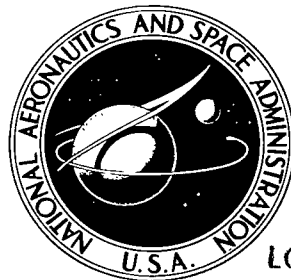


NASA TECHNICAL NOTE

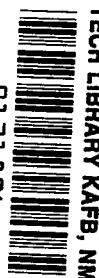


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A FORTRAN IV PROGRAM TO ESTIMATE THE OFF-DESIGN PERFORMANCE OF RADIAL-INFLOW TURBINES

by Carroll A. Todd and Samuel M. Futral, Jr.

Lewis Research Center

Cleveland, Ohio



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NATIONAL AERONAUTICS AND SPACE ADMINISTRATION

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ABSTRACT

This program computes conventional performance parameters at required combinations of speeds and flow rates. Necessary input information consists of flow areas, diameters, blade angles, and an estimate of design point performance. The flow equations and corresponding computer listings are given. A set of example calculations is included.

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SUMMARY

A FORTRAN IV computer program for off-design performance of a radial-inflow turbine is presented. The thermodynamic equations used and the corresponding computer listings are given. Use of the program requires as input information the turbine flow areas, diameters, and blade angles. An estimate of design point performance is also necessary. The output consists of conventional performance parameters at specified flow conditions and speeds. A complete set of calculations for an example turbine is included.

INTRODUCTION

A procedure for predicting the off-design performance of radial-inflow turbines was described by Futral and Wasserbauer in reference 1. It is possible to use this procedure for hand calculations, but it is lengthy and contains several iterations. For this reason a digital computer program analog has been used. The computer program used at Lewis Research Center is given here, with the explanations necessary for its use. The thermodynamic equations are also given. These are the equations of reference 1, except for an improvement in the calculation of rotor incidence loss and the calculation of additional performance parameters.

The program has been used in two ways. (1) For an existing turbine, the performance at design operating point may be known, while information on off-design performance is lacking. In this case, the program provides a convenient way to estimate off-design performance without making actual tests.

(2) The program is also useful as a design guide. For a proposed turbine design, the design point performance can usually be closely estimated as part of the design process. However, if the off-design operation is of sufficient importance, modifications in the design may be considered. In this case, the effect of a series as small design

changes on performance may be studied by repeated use of the program.

In addition to the engineering equations and the FORTRAN IV program, this report includes the input and output listings for an example turbine problem.

ENGINEERING ANALYSIS

The analysis consists of a one-dimensional solution of flow conditions along the mean streamline in the turbine, using perfect gas relations throughout. Equations were written in a step-by-step fashion, beginning at the turbine inlet. The analysis is written for subsonic flow only since stator choking is not expected. Symbols used in the engineering equations are given in appendix A, and the equations are listed in appendix B. Figure 1 shows the location of station subscripts in the turbine.

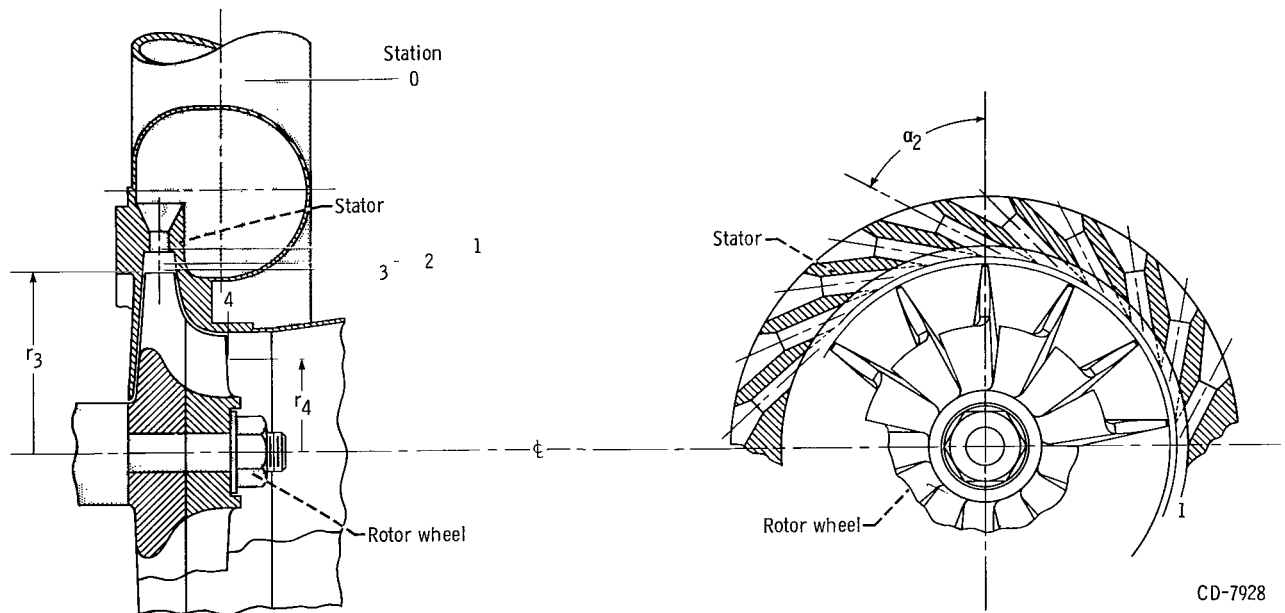


Figure 1. - Turbine stator and rotor.

CD-7928

Required Information

The geometric features of the turbine which must be specified consist of flow areas, blade angles, and major dimensions. The flow areas needed are those at the turbine inlet, stator exit, rotor inlet, and rotor exit. The turbine inlet area is normal to the flow, upstream of the scroll. The stator exit area and the rotor entrance area are surfaces of circular cylinders, with elements parallel to the turbine axis. The rotor exit area is an annular area, perpendicular to the turbine axis. The stator exit area is located just inside the blade row and the rotor inlet and exit areas are calculated at locations just outside the blade rows.

The blade angles at the stator exit and rotor exit must also be specified. These are angles at the mean streamline.

Diameters needed are the stator exit diameter, rotor inlet tip diameter, and the mean diameter at the rotor exit. The axial blade length at the rotor tip is also necessary.

The required inlet conditions of the working fluid are the total temperature and total pressure at the turbine inlet. The gas properties needed are the specific gas constant, the specific heat at constant pressure, and the specific heat ratio.

Input information which applies to the example turbine is as follows:

| | |
|--|----------|
| Turbine inlet area, sq ft | 0. 12989 |
| Stator inlet area, sq ft | 0. 04078 |
| Rotor entrance area, sq ft | 0. 04175 |
| Rotor exit area, sq ft | 0. 04719 |
| Stator exit diameter, sq ft | 0. 42650 |
| Rotor inlet diameter, sq ft | 0. 41417 |
| Stator exit axial dimension, ft | 0. 03208 |
| Stator exit angle, deg | 72. 47 |
| Rotor exit angle, deg | -56. 85 |
| Number of blades at rotor tip | 22 |
| Turbine inlet total pressure, °R | 2060 |

The thermodynamic properties of the working fluid are as follows:

| | |
|---|----------|
| Specific gas constant | 18. 437 |
| Specific heat ratio | 5/3 |
| Constant pressure specific heat | 0. 05924 |

The design point quantities, used in determination of frictional coefficients, are the following:

| | |
|---|--------|
| Mass flow rate, lb/sec | 0.7484 |
| Total efficiency | 0.897 |
| Wheel speed, rpm | 36 000 |
| Total pressure ratio across stator | 0.98 |
| Total pressure ratio across turbine | 1.7397 |

Operation of the Program

In solving the equations in appendix B, the computer uses the equations in essentially the order listed. Solutions for the quantities $(V/V_{cr})_0$, $(V/V_{cr})_2$, and $(V/V_{cr})_4$ are accomplished by means of the Newton-Ralphson method since these quantities are implicitly defined.

For operation of the program, the wheel speeds of interest are specified in advance. The initial value for $(V/V_{cr})_1$, the velocity ratio at the stator exit, as well as its increment size are also specified. The final value for this quantity will not be known and is determined by the computer for each wheel speed. For any given wheel speed, repeated increases in the value of $(V/V_{cr})_1$ will eventually result in a flow rate which the turbine cannot pass. Mathematically, this condition is evident in that no solution exists for $(W/W_{cr})_4$, the relative velocity ratio at the rotor exit. When this occurs, the maximum value for $(V/V_{cr})_1$ is computed to five significant figures. The computer prints out a complete set of answers for each value of $(V/V_{cr})_1$ for any given wheel speed. When one wheel speed is finished, the computer goes on to the next speed until all specified speeds have been used.

In this discussion, it is always assumed that the temperature and pressure of the working fluid entering the turbine are constant. This has fulfilled all needs thus far. However, if the program should be required to operate in some other manner, the amount of additional instructions would not be great.

Loss Coefficients

Calculation of rotor incidence loss makes use of the angle ϕ in equation (B17). This is the optimum incidence angle and is that incidence angle associated with minimum rotor incidence loss. The calculation of ϕ follows the method of Stanitz (ref. 2), developed to account for "slip" in centrifugal compressors having radial blading.

The loss coefficient k_1 applies to the stator and must be evaluated before the main program is used. The determination of k_1 involves simultaneous solution of equations (B1) to (B5) using the design value of mass flow rate. For this purpose, the total pressure ratio across the stator p_1'/p_0' is determined from design data. A separate routine is provided for this solution.

The determination of the loss coefficients m and k were presented as a graphical procedure in reference 1. In the computer determination, this is done by means of a search routine. By repeated calculations, the computer finds a pair of values for m and k which yield design values for mass flow rate, efficiency, and pressure ratio. Design speed and inlet conditions are specified.

Output Information

A sample of the computer output is shown for the example turbine. To assist in reading this, a list of machine output symbols in terms of engineering symbols is included in the section on program output. The information contained in the machine output may be classified under several headings as follows:

- (1) Input information - For each speed, the following quantities are printed out:
 - (a) Design speed
 - (b) Percent design speed
 - (c) Fluid inlet temperature and pressure
 - (d) Turbine dimensions and angles
 - (e) Fluid constants
 - (f) Number of rotor blades (at tip)
- (2) Performance parameters
 - (a) Efficiencies
 - (b) Blade-jet speed ratio
 - (c) Pressure ratios
 - (d) Specific work
 - (e) Ideal work
- (3) Internal variables - Internal variables are listed for each station throughout the turbine as follows:
 - (a) Pressures
 - (b) Temperatures
 - (c) Velocity components
- (4) Additional quantities - Certain ratios have been found useful and are included in the print-out, such as the following:

- (a) $wN\epsilon/\delta$
- (b) $N/\sqrt{T_0}$
- (c) $w\sqrt{T_0}/p'_{in}$
- (d) Γ/p'_{in}
- (e) Losses calculated on a pound mass basis denoted by L with appropriate subscript
- (f) Fractional losses denoted by $\Delta\eta$ where the loss has been divided by the ideal work
- (g) Losses in efficiency, given in equation (B31) as η_E , η_R , and η_I , which are useful in studying loss distributions in the turbine

THE FORTRAN IV PROGRAM

In this section are presented the FORTRAN IV routines, input and output considerations, and the internal documentation. The program operates in three distinct modes, each depending on the type of information available or the desired output. In mode one operation, the loss coefficients are assumed known. Off-design calculations are made by means of changes in speed and changes in the value of $(V/V_{cr})_1$. This mode furnishes performance results, once the loss coefficients are known. In mode two operation, sets of loss coefficients are entered and calculations are made at design values of speed and $(V/V_{cr})_1$. This mode is used for studying the results of variations in frictional effects. In mode three, the program determines the loss coefficients and produces one complete set of calculations representing design conditions of the turbine. This set of calculations verifies the determination of the loss coefficients.

Program Input

Actual input for the example with card format is presented in this section, followed by the definitions of the terms used on the cards.

| TITLE | | | | | | | | | | | | | | | | | PROJECT NUMBER | | | | | | | | | | | | | | | | | ANALYST | | | | | | | | | | | | | | | | | SHEET _____ OF _____ | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
|-------------------|---------|---|---|---|---|---|---|---|----|----|----|----|----|----|----|----|----------------|-------------------|----|----|----|----|----|----|----|----|----|----|----|----|----|----|----|---------|----------------|----|----|----|----|----|----|----|----|----|----|----|----|----|----|----|----------------------|----------|----|----|----|----|----|----|----|----|----|----|----|----|----|----|----|----|---------|----|----|----|----|----|----|----|----|----|----|--|--|--|--|--|--|---------|--|--|--|--|--|--|--|--|--|--|--|--|--|--|--|--|--------|--|--|--|--|--|--|--|--|--|--|--|--|--|--|--|--|
| STATEMENT NUMBER | CONT | | | | | | | | | | | | | | | | | FORTRAN STATEMENT | | | | | | | | | | | | | | | | | IDENTIFICATION | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 | 11 | 12 | 13 | 14 | 15 | 16 | 17 | 18 | 19 | 20 | 21 | 22 | 23 | 24 | 25 | 26 | 27 | 28 | 29 | 30 | 31 | 32 | 33 | 34 | 35 | 36 | 37 | 38 | 39 | 40 | 41 | 42 | 43 | 44 | 45 | 46 | 47 | 48 | 49 | 50 | 51 | 52 | 53 | 54 | 55 | 56 | 57 | 58 | 59 | 60 | 61 | 62 | 63 | 64 | 65 | 66 | 67 | 68 | 69 | 70 | 71 | 72 | 73 | 74 | 75 | 76 | 77 | 78 | 79 | 80 | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| MODE THREE FORMAT | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| PREF | WREF | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| .98667 | .798 | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| A0 | A1 | | | | | | | | | | | | | | | | | A3 | | | | | | | | | | | | | | | | | A4 | | | | | | | | | | | | | | | | | D1 | | | | | | | | | | | | | | | | | D3 | | | | | | | | | | | | | | | | | XL | | | | | | | | | | | | | | | | | AL1 | | | | | | | | | | | | | | | | |
| .05548 | .04078 | | | | | | | | | | | | | | | | | .04175 | | | | | | | | | | | | | | | | | .04718 | | | | | | | | | | | | | | | | | .4265 | | | | | | | | | | | | | | | | | .414166 | | | | | | | | | | | | | | | | | .032083 | | | | | | | | | | | | | | | | | 72.13 | | | | | | | | | | | | | | | | |
| B4 | ZBR | | | | | | | | | | | | | | | | | PG | | | | | | | | | | | | | | | | | TG | | | | | | | | | | | | | | | | | V3 | | | | | | | | | | | | | | | | | V4 | | | | | | | | | | | | | | | | | G | | | | | | | | | | | | | | | | | R | | | | | | | | | | | | | | | | |
| -5.6.55 | 22. | | | | | | | | | | | | | | | | | 3.588.12 | | | | | | | | | | | | | | | | | 2060. | | | | | | | | | | | | | | | | | 780.6838 | | | | | | | | | | | | | | | | | 443.245 | | | | | | | | | | | | | | | | | 1.6667 | | | | | | | | | | | | | | | | | 18.437 | | | | | | | | | | | | | | | | |
| CP | VSTART | | | | | | | | | | | | | | | | | DELV | | | | | | | | | | | | | | | | | VEND | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| .059242 | .2 | | | | | | | | | | | | | | | | | .05 | | | | | | | | | | | | | | | | | .9 | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| MODE | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| 3 | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| P.D.S.G.N | E.T.A.D | | | | | | | | | | | | | | | | | P.E | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| 1.734 | .913 | | | | | | | | | | | | | | | | | 22 | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |

The definitions of the terms used on the cards are as follows:

Card 1:

PREF estimate of p'_1/p'_0 to determine k_1
WREF estimate of W to determine k_1

Card 2:

AO area of turbine inlet, A_0 , sq ft
A1 area upstream of stator exit, A_1 , sq ft
A3 area upstream of rotor inlet, A_3 , sq ft
A4 area downstream of stator exit, A_4 , sq ft
D1 diameter upstream of stator exit, D_1 , ft
D3 diameter upstream of rotor inlet, D_3 , ft
XL axial distance, l , ft
AL1 absolute gas angle, α_1 , deg

Card 3:

B4 free stream gas flow angle, β_4 , deg
ZBR number of rotor blades at tip of rotor, Z
PO inlet pressure, p'_0 , lb/sq ft
TO inlet temperature, T'_0 , $^{\circ}\text{R}$
U3 design speed upstream of rotor inlet, U_3 , ft/sec
U4 design speed downstream of rotor exit, U_4 , ft/sec
G ratio of specific heat at constant pressure to specific heat at constant volume, γ
R specific gas constant, R , ft-lb/(lb)($^{\circ}\text{R}$)

Card 4:


CP specific heat at constant pressure, c_p , Btu/(lb)($^{\circ}\text{R}$)
VSTART starting value of $(V/V_{cr})_1$
DELV incremental value of $(V/V_{cr})_1$
VEND end value of $(V/V_{cr})_1$

Card 5:
 MODE if MODE = 1, a performance run varying percent speed
 if MODE = 2, manual determination of m and k
 if MODE = 3, automatic determination of m and k

| | | | |
|----------|--|---|------------|
| Card 6: | | } | MODE ONE |
| XM | loss coefficient | | |
| XK | loss coefficients | | |
| Card 7+: | | } | MODE TWO |
| PERCT | percent of design speed | | |
| Card 6+: | | } | MODE THREE |
| XM | loss coefficient | | |
| XK | loss coefficient | | |
| Card 6: | | } | MODE THREE |
| PDSGN | specified p'_0/p'_4 or p'_0/p'_4 | | |
| ETAD | specified η_t or η_s | | |
| P | if $p = 1$, p'_0/p'_4 is specified if $p = 2$, p'_0/p'_4 is specified | | |
| E | if $E = 1$, η_s is specified if $E = 2$, η_t is specified | | |

Description of the Output

A list of the machine output symbols, internal program variables, and their corresponding engineering symbols is given here. Refer to appendix A for the physical interpretation. Following this are sample output listings showing the results of calculations for modes one, two, and three using the input data given previously. It should be noted that each block of output is for a specific value of $(V/V_{cr})_1$.



Machine Output Symbols and Definitions

| | |
|----------|----------------|
| ALPHA2 | α_2 |
| BETA3 | β_3 |
| DEE | $\Delta\eta_E$ |
| DEI | $\Delta\eta_I$ |
| DELTA | δ |
| DER | $\Delta\eta_R$ |
| DES | $\Delta\eta_S$ |
| EQ-SPEED | N_{eq} |
| ETA-E | η_E |
| ETA-I | η_I |
| ETAR | η_R |
| ETA-S | η_S |
| ETA-T | η_t |
| LE | L_E |
| LI | L_I |
| NOT | $N/\sqrt{T_0}$ |
| NU | ν |
| PO, /P4 | p'_0/p_4 |
| PO, /P4, | p'_0/p'_4 |

| | |
|------------|-----------------------------|
| P1, /PO, | p'_1/p'_0 |
| P2, /PO, | p'_2/p'_0 |
| P2, /P1, | p'_2/p'_1 |
| P4/PO, | p_4/p'_0 |
| P4/P4ID | $p''_4/p''_4, id$ |
| P4/P4, | p_4/p'_4 |
| P4, /PO, | p'_4/p'_0 |
| PHI | φ |
| PV/PVCR)O | $(\rho V/\rho' V_{cr})_0$ |
| PV/PVCR)1 | $(\rho V/\rho' V_{cr})_1$ |
| PV/PVCR)2 | $(\rho V/\rho' V_{cr})_2$ |
| PVX/PVCR)3 | $(\rho V_x/\rho' V_{cr})_3$ |
| PW/PWCR)4 | $(\rho W/\rho'' W_{cr})_4$ |
| R2 | r_2 |
| RHO)O | ρ'_0 |
| RHO-VCR)O | $(\rho' V_{cr})_0$ |
| ROTOR-LOSS | L_R |
| SMALL-W | w |
| SPEED | N |
| SPEED-PAR | wN/δ |

| | |
|--------------|------------------|
| STATOR-LOSS | L_S |
| $T_{,,/T,}3$ | $(T''/T')_3$ |
| $T3,,$ | T'_3 |
| $T4,$ | T'_4 |
| $T4,/T4,,$ | $(T'/T'')_4$ |
| $T4,,/T3,,$ | T''_4/T''_3 |
| THETA-CR | θ_{cr} |
| TOP | Γ/p'_0 |
| V3 | V_3 |
| V4 | V_4 |
| $V/V_{CR}0$ | $(V/V_{cr})_0$ |
| $V/V_{CR}1$ | $(V/V_{cr})_1$ |
| $V/V_{CR}2$ | $(V/V_{cr})_2$ |
| $V/V_{CR}3$ | $(V/V_{cr})_3$ |
| $V_{CR}0$ | $V_{cr,0}$ |
| $V_{CR}4$ | $V_{cr,4}$ |
| VU3 | $V_{u,3}$ |
| VU4 | $V_{u,4}$ |
| $VU/V_{CR}3$ | $(V_u/V_{cr})_3$ |
| VX3 | $V_{x,3}$ |

| | |
|----------|---------------------|
| VX/VCR)3 | $(v_x/v_{cr})_3$ |
| W1 | $\Delta h'$ |
| W3 | w_3 |
| W4 | w_4 |
| W/WCR)3 | $(w/w_{cr})_3$ |
| W/WCR)4 | $(w/w_{cr})_4$ |
| WCR3 | $w_{cr, 3}$ |
| WCR4 | $w_{cr, 4}$ |
| WEQ | w_{eq} |
| WID | $\Delta h'_{id, s}$ |
| W, ID | $\Delta h'_{id, t}$ |
| WTOP | $w\sqrt{T'_0/p'_0}$ |

Mode One Sample Output

OFF DESIGN PERFORMANCE OF A RADIAL INFLOW TURBINE

| | | | | | | | | | | |
|------------------|----|----------|----|--------------|-----|--------------|-----|--------------|----|----------|
| TURBINE GEOMETRY | AQ | 0.125887 | A1 | 0.407800E-01 | A3 | 0.417450E-01 | A4 | 0.471880E-01 | D1 | 0.426500 |
| | D3 | 0.414166 | L | 0.320830E-01 | AL1 | 72.47000 | B4 | -56.85000 | Z | 22.00000 |
| INLET CONDITIONS | PD | 3600.000 | TO | 2060.000 | | | | | | |
| DESIGN SPEEDS | U3 | 780.6838 | U4 | 443.2450 | | | | | | |
| FLUID PROPERTIES | G | 1.666700 | R | 18.43700 | CP | 0.592420E-01 | EPS | 0.910949 | | |

MODE 1. PERFORMANCE RUN VARYING PERCENT SPEED

| | |
|----------------|--------------|
| M | 3.592625 |
| K | 0.754039E-01 |
| K1 | 0.763553E-01 |
| PERCENT | 100.0000 |
| U3 | 780.6838 |
| U4 | 443.2450 |
| DESIGN PL./PO. | 0.980000 |
| DESIGN SMALL-W | 0.748400 |

| | | | | | | | |
|------------|--------------|-------------|--------------|------------|--------------|-----------|--------------|
| V/VCR)1 | 0.200000 | P2./PO. | 0.998079 | V3 | 253.5031 | WCR3 | 1258.994 |
| VCR)0 | 1235.911 | P2./P1. | 1.000023 | BETA3 | -82.29354 | W4 | 132.7295 |
| RHO)0 | 0.947862E-01 | ALPHA2 | 73.31849 | T.,/T.,J3 | 1.037702 | WCR4 | 1217.305 |
| RHO-VCR)0 | 117.1473 | R2 | 0.212270 | T3., | 2137.666 | VU4 | 332.1319 |
| PV/PVCR)1 | 0.157008 | VU/VCR)3 | 0.196464 | T4.,/T3., | 0.934870 | T4.,/T4., | 1.016578 |
| SMALL-W | 0.283144 | VX/VCR)3 | 0.589386E-01 | W/WCR)3 | 0.431123 | P4./PO. | 0.885221 |
| PV/PVCR)0 | 0.186084E-01 | V/VCR)3 | 0.205114 | PVX/PVCR)3 | 0.580112E-01 | VCR4 | 1227.323 |
| V/VCR)0 | 0.166108E-01 | VU3 | 242.8122 | P4/P4ID | 0.918390 | V4 | 339.9744 |
| V/VCR)2 | 0.200089 | W3 | 542.7817 | PW/PWCR)4 | 0.108550 | P4/P4. | 0.952730 |
| PV/PVCR)2 | 0.197093 | VX3 | 72.84285 | W/WCR)4 | 0.109036 | P4/PO. | 0.843376 |
| W1 | 1.651284 | ETA-E | 0.497547 | W, ID | 5.808875 | T4., | 2031.472 |
| WID | 8.38581 | ETA-T | 0.291155 | ETA-S | 0.210396 | WEQ | 0.183813 |
| ETA-I | 0.778959 | LE | 2.308287 | DEE | 0.287151 | LI | 2.262151 |
| ROTOR-LOSS | 1.689182 | STATOR-LOSS | 0.928868E-01 | DER | 0.210134 | DES | 0.115551E-01 |
| SPFED | 36000.00 | EQ-SPEED | 29695.14 | NU | 1.730507 | THETA-CR | 1.469718 |
| DELTA | 1.701146 | SPEED-PAR | 5991.953 | W1-EQ | 1.150754 | PO./P4. | 1.129662 |
| NOT | 793.1747 | WTOP | 0.514045 | TOP | 0.474460E-01 | DEI | 0.281412 |
| PO./P4 | 1.185710 | PHI | -43.96986 | P1./PO. | 0.998056 | ETA-R | 1.066110 |
| NSP | 120.8699 | | | | | | |

| | | | | | | | |
|------------|--------------|-------------|--------------|------------|--------------|-----------|--------------|
| V/VCR)1 | 0.250000 | P2./PO. | 0.996982 | V3 | 316.8903 | WCR3 | 1249.548 |
| VCR)0 | 1235.911 | P2./P1. | 1.000035 | BETA3 | -79.19788 | W4 | 167.5706 |
| RHO)0 | 0.947862E-01 | ALPHA2 | 73.31857 | T.,/T.,J3 | 1.022189 | WCR4 | 1207.532 |
| RHO-VCR)0 | 117.1473 | R2 | 0.212270 | T3., | 2105.710 | VU4 | 302.9650 |
| PV/PVCR)1 | 0.244164 | VU/VCR)3 | 0.245580 | T4.,/T3., | 0.933882 | T4.,/T4., | 1.012364 |
| SMALL-W | 0.350528 | VX/VCR)3 | 0.737055E-01 | W/WCR)3 | 0.388769 | P4./PO. | 0.847779 |
| PV/PVCR)0 | 0.230369E-01 | V/VCR)3 | 0.256402 | PVX/PVCR)3 | 0.718959E-01 | VCR4 | 1214.974 |
| V/VCR)0 | 0.230415E-01 | VU3 | 303.5152 | P4/P4ID | 0.926179 | V4 | 316.5768 |
| V/VCR)2 | 0.250111 | W3 | 485.7858 | PW/PWCR)4 | 0.137770 | P4/P4. | 0.958118 |
| PV/PVCR)2 | 0.244267 | VX3 | 91.09343 | W/WCR)4 | 0.138771 | P4/PO. | 0.812273 |
| W1 | 4.100490 | ETA-E | 0.626467 | W, ID | 7.800876 | T4., | 1990.797 |
| WID | 5.739316 | ETA-T | 0.525645 | ETA-S | 0.421024 | WFQ | 0.227558 |
| ETA-I | 0.839335 | LE | 2.000868 | DEE | 0.205442 | LI | 2.073232 |
| ROTOR-LOSS | 1.428626 | STATOR-LOSS | 0.145112 | DER | 0.146686 | DES | 0.148996E-01 |
| SPFED | 36000.00 | EQ-SPEED | 29695.14 | NU | 1.117917 | THETA-CR | 1.469718 |
| DELTA | 1.701146 | SPEED-PAR | 7417.945 | W1-EQ | 2.789984 | PO./P4. | 1.179552 |
| NOT | 793.1747 | WTOP | 0.636380 | TOP | 0.142408 | DEI | 0.212872 |
| PO./P4 | 1.221113 | PHI | -37.64624 | P1./PO. | 0.996947 | ETA-R | 1.044782 |

Mode Two Sample Output

OFF DESIGN PERFORMANCE OF A RADIAL INFLOW TURBINE

| | | | | | | | | | | |
|------------------|----|----------|----|--------------|-----|--------------|-----|--------------|----|----------|
| TURBINE GEOMETRY | AU | 0.129887 | AI | 0.407300E-01 | A3 | 0.417450E-01 | A4 | 0.471880E-01 | D1 | 0.426500 |
| | D3 | 0.414160 | L | 0.320330E-01 | AL1 | 72.47000 | B4 | -56.85000 | Z | 22.00000 |
| INLET CONDITIONS | PU | 3600.000 | TU | 2060.000 | | | | | | |
| DESIGN SPEEDS | U3 | 780.8838 | U4 | 443.2450 | | | | | | |
| FLUID PROPERTIES | G | 1.666700 | R | 18.43700 | CP | 0.592420E-01 | EPS | 0.910949 | | |

MODE 2: MANUAL DETERMINATION OF M AND K
M 3.000000
K 0.500000E-01
K1 0.763553E-01
PERCENT 100.0000
U3 780.8838
U4 443.2450
DESIGN P1, /P0, 0.960000
DESIGN SMALL-W 0.748400

| | | | | | | | |
|------------|--------------|-------------|--------------|--------------|--------------|------------|--------------|
| V/VCK11 | 0.200000 | P2, /P0, | 0.998695 | V3 | 256.3198 | WCR3 | 1258.537 |
| VCR10 | 1235.911 | P2, /P1, | 1.000640 | BETA3 | -82.25358 | W4 | 130.5060 |
| KHU10 | 0.947802E-01 | ALPHA2 | 73.51138 | T, /T, 13 | 1.036950 | WCR4 | 1216.832 |
| RHU-VCK11 | 117.1473 | K2 | 0.212058 | T3, , | 2136.116 | VU4 | 333.9932 |
| PV/PVCK11 | 0.197000 | VU/VCK13 | 0.193847 | T4, , /T3, , | 0.934823 | T4, /T4, , | 1.016820 |
| SMALL-W | 0.283144 | VX/VCK13 | 0.589232E-01 | W/ WCR13 | 0.428959 | P4, /P3, , | 0.900118 |
| PV/PVCK10 | 0.186084E-01 | V/VCK13 | 0.207393 | PVX/PVCK13 | 0.579754E-01 | VCR4 | 1227.023 |
| V/VCK10 | 0.186108E-01 | VU3 | 245.7571 | P4/ P4ID | 0.934413 | V4 | 341.5368 |
| V/VCK12 | 0.202516 | W3 | 539.8610 | PN/ P WCR14 | 0.106788 | P4/ P4, | 0.952278 |
| PV/PVCK12 | 0.199409 | VX3 | 72.82388 | W/ WCR14 | 0.107251 | P4/ PD, | 0.857153 |
| W1 | 1.750150 | ETA-E | 0.555911 | WID | 5.030349 | T4, | 2030.478 |
| W10 | 7.296778 | ETA-T | 0.347920 | ETA-S | 0.239853 | WEQ | 0.183813 |
| ETA-I | 0.805162 | LE | 2.329552 | DER | 0.319258 | LI | 2.233187 |
| KUTUK-LUSS | 0.924090 | STATUR-LUSS | 0.630830E-01 | DER | 0.126645 | DES | 0.864533E-02 |
| SPEED | 36000.00 | EQ-SPEED | 29695.14 | VJ | 1.291541 | THETA-CR | 1.459718 |
| DELTA | 1.701146 | SPEED-PAR | 5991.953 | W1-EQ | 1.190812 | PD, /P4, | 1.110965 |
| NU1 | 793.1747 | WTOP | 0.514045 | TJP | 0.490976E-01 | DEI | 0.306051 |
| PU, /P4 | 1.186640 | PH1 | -43.97732 | PL, /PJ, | 0.998056 | ETA-R | 1.184420 |
| NSP | 133.5119 | | | | | | |

| | | | | | | | |
|------------|--------------|-------------|--------------|--------------|--------------|------------|--------------|
| V/VCK11 | 0.200000 | P2, /P0, | 0.997949 | V3 | 320.4066 | WCR3 | 1248.973 |
| VCR10 | 1235.911 | P2, /P1, | 1.001005 | BETA3 | -79.12022 | W4 | 165.0909 |
| KHU10 | 0.947802E-01 | ALPHA2 | 73.51138 | T, /T, 13 | 1.021249 | WCR4 | 1206.937 |
| RHU-VCK11 | 117.1473 | K2 | 0.212058 | T3, , | 2103.772 | VU4 | 305.0409 |
| PV/PVCK11 | 0.244164 | VU/VCK13 | 0.243558 | T4, , /T3, , | 0.933821 | T4, /T4, , | 1.012692 |
| SMALL-W | 0.350528 | VX/VCK13 | 0.736753E-01 | W/ WCR13 | 0.386049 | P4, /P3, , | 0.850185 |
| PV/PVCK10 | 0.230369E-01 | V/VCK13 | 0.259247 | PVX/PVCK13 | 0.718263E-01 | VCR4 | 1214.572 |
| V/VCK10 | 0.230415E-01 | VU3 | 307.1957 | P4/ P4ID | 0.940376 | V4 | 318.1267 |
| V/VCK12 | 0.253144 | W3 | 482.1640 | PN/ P WCR14 | 0.135826 | P4/ P4, | 0.957672 |
| PV/PVCK12 | 0.247086 | VX3 | 91.05610 | W/ WCR14 | 0.136785 | P4/ PD, | 0.823775 |
| W1 | 4.180503 | ETA-E | 0.588837 | WID | 7.135124 | T4, | 1989.480 |
| W10 | 9.105924 | ETA-T | 0.589524 | ETA-S | 0.458877 | WEQ | 0.227558 |
| ETA-I | 0.904348 | LE | 2.021145 | DER | 0.221959 | LI | 2.035274 |
| RUTUK-LUSS | 0.778077 | STATUR-LUSS | 0.905512E-01 | DER | 0.854474E-01 | DES | 0.108228E-01 |
| SPEED | 36000.00 | EQ-SPEED | 29695.14 | VJ | 1.156144 | THETA-CR | 1.459718 |
| DELTA | 1.701146 | SPEED-PAR | 7417.945 | W1-EQ | 2.843064 | PD, /P4, | 1.162541 |
| NU1 | 793.1747 | WTOP | 0.535380 | TJP | 0.145117 | DEI | 0.223511 |
| PU, /P4 | 1.213924 | PH1 | -37.65760 | PL, /PJ, | 0.996947 | ETA-R | 1.126307 |

Mode Three Sample Output

OFF DESIGN PERFORMANCE OF A RADIAL INFLOW TURBINE

| | | | | | | | | | | |
|------------------|----|----------|----|--------------|-----|--------------|-----|--------------|----|----------|
| TURBINE GEOMETRY | AD | 0.125887 | A1 | 0.407800E-01 | A3 | 0.417450E-01 | A4 | 0.471880E-01 | D1 | 0.426500 |
| | D3 | 0.414166 | L | 0.320830E-01 | AL1 | 72.47000 | B4 | -56.85000 | Z | 22.00000 |
| INLET CONDITIONS | PD | 3600.000 | TD | 2060.000 | | | | | | |
| DESIGN SPEEDS | U3 | 780.6838 | U4 | 443.2450 | | | | | | |
| FLUID PROPERTIES | G | 1.666700 | R | 18.43700 | CP | 0.592420E-01 | EPS | 0.910949 | | |

MODE 3. AUTOMATIC DETERMINATION OF M AND K

| | |
|----------------|--------------|
| M | 3.947835 |
| K | 0.683671E-01 |
| K1 | 0.763553E-01 |
| PFRCENT | 100.0000 |
| U3 | 780.6838 |
| U4 | 443.2450 |
| DESIGN P1./PD, | 0.980000 |
| DESIGN SMALL-W | 0.748400 |
| DESIGN PD./P4, | 1.740000 |
| DESIGN ETA-T | 0.917000 |

| | | | | | | | |
|------------|--------------|-------------|-----------|------------|--------------|-----------|--------------|
| V/VCR)1 | C.615818 | P2./PD, | 0.981934 | V3 | 783.3564 | WCR3 | 1177.725 |
| VCR)0 | 1235.911 | P2./P1, | 1.001973 | BETA3 | -7.711356 | W4 | 444.0692 |
| RHO)0 | 0.947862E-01 | ALPHA2 | 73.38271 | T.,/T,)3 | 0.908058 | WCR4 | 1133.050 |
| RHO-VCR)0 | 117.1473 | R2 | 0.212199 | T3,, | 1870.599 | VUJ4 | 71.49717 |
| PV/PVCR)1 | 0.530352 | VU/VCR)3 | 0.606932 | T4.,/T3,, | 0.925572 | T4.,/T4,, | 0.974083 |
| SMALL-W | 0.748445 | VX/VCR)3 | 0.182683 | W/WCR)3 | 0.193457 | P4./PD, | 0.575019 |
| PV/PVCR)0 | 0.491883E-01 | V/VCR)3 | 0.633829 | PVX/PVCR)3 | 0.155864 | VCR4 | 1118.271 |
| V/VCR)0 | 0.492331E-01 | VU3 | 750.1139 | P4/P4)D | 0.965595 | V4 | 253.2051 |
| V/VCR)2 | 0.618131 | W3 | 227.8395 | PW/PWCR)4 | 0.369567 | P4/P4, | 0.968264 |
| PV/PVCR)2 | 0.531715 | VX3 | 225.7794 | W/WCR)4 | 0.391974 | P4/PD, | 0.556771 |
| W1 | 22.12419 | ETA-E | 0.918337 | W,)D | 24.23218 | T4, | 1686.502 |
| WID | 25.48582 | ETA-T | 0.913009 | ETA-S | 0.868098 | WEQ | 0.485880 |
| FTA-I | 0.919463 | LE | 1.280390 | DEE | 0.502393E-01 | LI | 0.286848E-01 |
| ROTOR-LOSS | 1.342740 | STATOR-LOSS | 0.901912 | DER | 0.526858E-01 | DES | 0.314650E-01 |
| SPFFD | 36000.00 | EQ-SPEED | 29695.14 | VU | 0.691074 | THETA-CR | 1.469718 |
| DELTA | 1.701146 | SPEED-PAR | 15838.75 | W1-EQ | 15.05336 | PD./P4, | 1.739072 |
| NNT | 793.1747 | WTOP | 1.358794 | TOP | 1.640600 | DEI | 0.112552E-02 |
| PD./P4 | 1.796072 | PHI | -17.28718 | P1./PD, | 0.980000 | FTA-R | 0.969702 |
| NSP | 75.74157 | | | | | | |

Main Program Listing

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C A FORTRAN IV COMPUTER ROUTINE TO PREDICT OFF DESIGN PERFORMANCE
C OF A RADIAL INFLOW TURBINE
C
  SUB(X,G)=X*(1.-(G-1.)/(G+1.)*X*X)**(1./(G-1.))
  DATA AC/32.1739/,PI/3.14159/,XJ/778.16/,F1/.01745/
  DATA TOL/.000001/,XNK/9./,XMN/9./
  COMMON /GFK/PREF,WRFF,G,AC,A1,AL1,TO,R,PO
  1, PDSGN,CP,G1,G2,G3,ETAD,U3,U4,VU3,B4,T4T3,WCR3,TT3,W3WC3,B3,
  2 PHI,P2PP1P,P1PP0P
  INTEGER PTEST , ETEST
  RFAD(5,40C)PREF,WRFF
C
C INPUT TURBINE GEOMETRY-INLET CONDITIONS-SPEEDS-FLUID PROPERTIES
C
  1 READ(5,400) AO,A1,A3,A4,D1,D3,XL,AL1,B4,ZBR ,PG,TO,
  U3,U4,      G,R,CP,      VSTART,DELV1,VEND
  U31=U3
  U41=U4
  CALL GETK1(XK1,V1VC1Q)
C
  G1=G+1.
  G2=G-1.
  G2=G2/G1
  EPS=1.4/G*((G1/2.)**(G/G2)/1.8929)
  R1=D1/2.
  R3=C3/2.
C DETERMINE MODE OF OPERATION
C
  READ(5,401)MODE
C MODE=1, IMPLIES A PERFORMANCE RUN VARYING PERCENT SPEED
C MODE=2, IMPLIES INPUTTING VARIOUS VALUES OF M AND K AT DESIGN SPEED
C MODE=3, IMPLIES LETTING THE MACHINE DETERMINE THE VALUE OF M AND K
C
  30 GO TO (2,3,4),MODE
  2 READ(5,400)XM,XK
  5 READ(5,400) PFRCT
  V1VC1=VSTART
  DELV =DELV1
  GO TO 6
  3 READ(5,400) XM,XK
  PFRCT=100.
  V1VC1=VSTART
  DELV =DELV1
  GO TO 6
  4 RFAD(5,405) PDSGN,ETAD,PTEST,E TEST
405 FFORMAT(2F10.4,2I1)
  XK=1.E-8
  K=1
  V1VC1=V1VC1Q
  PFRCT=100.

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6 U3=U31*PERCT/100.
  U4=U41*PERCT/100.
C CASE CONSTANTS
  GO TO 40
50 VCR0=SQRT(2./G1*AC*TD*G*R)
  RHOOP=PO/R/TO
  PVCRO=VCR0*RHOOP
C STATOR ANALYSIS
  COSAL1=COS(AL1*F1)
  SINAL1=SIN(AL1*F1)
C
23 CONTINUE
  RHOVC1=SUB(V1VC1,G)
  Q1=2.*XK1*RHOVC1*A1*COSAL1/AO/G1
  Y=V1VC1
  X=Y/2.
  Q1=Q1/(1.-Y*Y)
10 P1PPOP=(1.-XK1*G3*(X*X+Y*Y)/(1.-G3*Y*Y))*G/G2
  SW=P1PPOP*RHOVC1*PVCRO*A1*COSAL1
  RHOVCO=SW/PVCRO/AO
  F=SUB(X,G)-RHOVCO
  Q=(F+RHOVCO)/X
  FP=Q-2.*X**2/G1*Q**(2.-G)+X*Q1*P1PPOP**(2.-G)
  X1=X-F/FP
  IF(ABS((X1-X)/X1).LT.TOL) GO TO 9
  X=X1
  GO TO 10
C
C STATOR TO ROTOR ANALYSIS
9 VQVCO=X1
  V2VC2=(XK1+1.)/(XK1+1.)*Y*Y-(XK1-XK1)/(XK1+1.)*X*X
  V2VC2=SQRT(V2VC2)
  R2VC2=SUB(V2VC2,G)
  P2PPOP=(1.-G3*XK1*(VQVCO**2+V2VC2**2)/(1.-G3*V2VC2**2))*G/G2
  P2PP1P=P2PPOP/P1PPOP
  TANAL2=2.*PI*R1*XL*KH2VC2*P2PP1P*SINAL1/(RHCVC1*A1*CCSAL1)
  AL2=ATAN(TANAL2)
  SINAL2=SIN(AL2)
  R2=R1*SINAL1/SINAL2
C
C ROTOR INLET ANALYSIS
C
  VLVC3=R2/R3*V2VC2*SINAL2
  X=V1VC1/2.
12 Q=R2/R3*RH2VC2*COS(AL2)
  F=Q*(1.-G3*(X*X+VUVC3**2))*(-1./G2)-X
  FP=2./G1*X*Q**2/(F+X)-1.
  X1=X-F/FP
  IF(ABS((X1-X)/X1).LT.TOL) GO TO 11
  X=X1
  GO TO 12
11 VXVC3=X1
  V3VC3=SQRT(VXVC3**2+VUVC3**2)
  VU3=VCR0*VUVC3
  VX3=VCR0*VXVC3
  V3=VCR0*V3VC3

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W3=(V3**2-2.*U3*VU3+U3**2)**.5
COSB3=VX3/W3
SINB3=(VU3-U3)/W3
B3=AR SIN(SINB3)
TT3=1.-G3*(2.*U3*VU3-U3**2)/VCR0**2
T3PP=TO*TT3
T4T3=1.-(U3**2-U4**2)/(2.*AC*XJ*CP*T3PP)
W3WC3=VXVC3/(SQRT(TT3)*COSB3)
WCR3=W3/W3WC3
RXVC3=SUB(V3VC3,G)*VXVC3/V3VC3
WX3=W3*COSB3
VU3OPT=U3*(1.-1.98/ZBR)
WU3OPT=VU3OPT-U3
PHI=ATAN(WU3OPT/WX3)
C
IF(MODE.NE.3)GO TO 300
CALL GFIM(XK,XM,1,ETAD,1)
300 CONTINUE
C ROTOR ANALYSIS
C
X=V1VC1/2.
Q=RHXVC3*A3/(A4*COS(B4*F1))/(TT3*T4T3)**(G1/G2/2.)
Q1=W3WC3**2*(XM*XK+SIN(B3-PHI)**2)/T4T3
KCWNT=0
15 Q2=1.-G3*X*X
KCWNT=KCWNT+1
Q3=XM*XK*X*X+Q1
F2=(1.-G3*(Q3)/Q2)**(G/G2)
F3=SUB(X,G)
F=Q/F2-F3
F2P=G/G2*F2**2*(1./G)*(-2.*G3**2*Q3/Q2**2*X-2.*G3*XM*XK*X/Q2)
F3P=-2.*X**2/G1*(F3/X)**(2.-G)+F3/X
FP=-Q/F2**2*F2P-F3P
X1=X-F/FP
C
C CHECK FOR CHOKING CONDITIONS
C
IF((X1.GT.1.).OR.(X1.LT.0.).OR.(KCWNT.GT.300))GO TO 13
IF(ABS((X1-X)/X1).LT.TOL)GO TO 14
X=X1
GO TO 15
14 P4P4ID=F2
RHWCR4=F3
W4WC4=X
WCR4=WCR3*SQRT(T4T3)
W4=WCR4*W4WC4
VU4=U4-W4*ABS(SIN(F1*B4))
WU4=W4*SIN(F1*B4)
T4T4=1.+G3*(U4**2+2.*U4*WU4)/WCR4**2
P4PPOP=P1PPOP*P4P4ID*(TT3*T4T3*T4T4)**(G/G2)*P2PP1P
SINB4=SIN(F1*B4)
COSB4=COS(F1*B4)
VCR4=WCR4*SQRT(T4T4)
V4=SQRT(W4**2*COSB4**2+(U4+W4*SINB4)**2)
P4P4P={1.-G3*(V4/VCR4)**2)**(G/G2)
P4POP=P4PPOP*P4P4P

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T4P=T4T4*T4T3*T3PP
THETCR=(VCR0/1019.46)**2
WK=VU3*U3-VU4*U4
WK=WK/AC/XJ
WK1=CP*(T0-T4P)
WIDP=CP*T0*(1.-P4POP**G2/G) 84
WID=CP*T0*(1.-P4POP**G2/G) 85
ETAT=WK/WIDP
ETAS=WK/WID
P3P4P=(1.-G3*(V3VC3**2-(VUVC3**2-(U3/VCR0)**2)**2)**G/G2) 86
P3P3P=(1.-G3*V3VC3**2)**G/G2) 87
P4PP3P=P4PP0P/P2PP0P
WRIDP=CP*T0*(1.-P4PP3P**G2/G)-(U3-VU3)**2/(2.*AC*XJ) 88
RCTLDS=XM*XK*(W3**2+W4**2)/(2.*AC*XJ)
VC=VCR0*V0VC0
V2=VCR0*V2VC2
STALOS=XK*(V0**2+V2**2)/2./AC/XJ
  CER=RDTLOS/WID
  CES=STALOS/WID
DEL=P0/2116.22
WEQ=SW*SQR(THETCR)/DEL*EPS 89
WIFQ=WK/THETCR
DEE=V4**2/(2.*AC*XJ*WID)
ETAE=ETAS+DEE
XLF=V4**2/2./XJ/AC
DEI=(.73*SIN(B3-PHI))**2/(2.*AC*XJ*WID) 90
XLI=DEI*WID
AL2=57.3*AL2
R3=57.3*B3
PHI=57.3*PHI
FTAI=ETAE+DEI
ETAR=ETAI+DEE
XN=60.*U3/PI/D3
EQSPD=XN/SQR(THETCR) 91
XNU=U3/SQR(2.*AC*XJ*WID) 92
SPPAR=SW*XN*FPS/DEL
PCPP4P=1./P4PP0P
XNPT=XN/SQR(T0) 93
WTOP=SW*SQR(T0)*144./P0 94
XTOP=SW*WK*3.5616*144./XN/P0*AC*XJ
P4=P4POP*P0
T4=T4P*(1.-G3*(V4/VCR4)**2)
RHO4=P4/R/T4
Q4=SW/RHO4
C0=U3/SQR(2.*AC*XJ*WIDP) 95
H4=XJ*WIDP
XNSP=XN*SQR(Q4)/H4**.75 96
SPPAR=SW*XN/DEL
G0D=7537.129*WK*SW/P0/XN*AC*XJ
PCPP4=1./P4POP
IF(MODE.NE.3) GO TO 20
EX=FTAS
IF(ETEF.F0.2) EX=ETAT
IF(K.EQ.6) GO TO 25
CALL (FTMIK,XM,2,EX,K) 106
GO TO 23

```

```

21 V1VC1=V1VC1+DELV
   IF(V1VC1.GT.VEND) GO TO 22
   GO TO 23
22 V1VC1=VSTART
   DELV=DELV1
   IF(MODE.EQ.1) GO TO 5
   IF(MODE.EQ.2) GO TO 3
   IF(MODE.EQ.3) GO TO 4
25 CONTINUE
   WRITE(6,509) XM,XK,XK1,PERCT,U3,U4,PREF,WREF
   IF(PTEST .EQ. 1) WRITE(6,529) PDSGN
   IF(PTEST .EQ. 2) WRITE(6,526) PDSGN
   IF(ETEST .EQ. 1) WRITE(6,527) ETAD
   IF(FTEST .EQ. 2) WRITE(6,528) ETAD
   GO TO 20
C CHKFC
13 IF(MODE.EQ.2) GO TO 20
   V1VC1=V1VC1-DELV
   DELV=DELV/2.
   IF(DELV.LT.5.F-4) GO TO (5,3),MODE
   GO TO 21
C
C
C          OUTPUT FOR SPEED V1VC1
20 CONTINUE
   WRITE(6,505)
505 FORMAT(1F9)
   WRITE(6,506) V1VC1,P2PPOP,V3,WCR3,VCRO,P2PP1P,B3,W4,RHOOP,
1AL2,TT3,WCR4,PVCR0,R2,T3PP,VU4,RHOVC1,VUVC3,T4T3,T4T4,SW,
2VXVC3,W3WC3,P4PPOP,RHOVC0,V3VC3,RHXVC3,VCRA
C
   WRITE(6,507) VOVC0,VU3,P4P4ID,V4,V2VC2,W3,RHWCR4,P4P4P,
1KH2VC2,VX3,W4WC4,P4POP,WK,ETA,EAE,WIDP,T4P,WID,ETAT,ETAS,WEQ,
2ETA1,XLF,DEF,XLI,ROTLOS,STALOS,DER,DES,XN,EQSPD,XNU,THETCR
C
   WRITE(6,508) DEL,SPPAR,W1EC,POPP4P,XNOT,WTOP,XTOP,DEI,
1POPP4,PHI,P1PPOP,ETAR ,XNSP
C
   GO TO 21
C          OUTPUT INPUT
40 WRITE(6,501) AQ,A1,A3,A4,D1,D3,XL,AL1,B4,ZBR, PD,TD,
1U31,U41,G,R ,CP ,EPS
   IF(MODE.EQ.1) WRITE(6,502)
   IF(MODE.EQ.2) WRITE(6,503)
   IF(MODE.EQ.3) GO TO 525
509 FORMAT(14X,1HM,G15.6/14X,1HK,G15.6/13X,2HK1,G15.6/7X,8H PERCENT,
1G15.6 /13X,2HU3,G15.6/13X,2HU4,G15.6/1X,14HDESIGN P1,/PO, G15.6 /
2 1X,14HDESIGN SMALL-W,G15.6 )
   WRITE(6,509) XM,XK,XK1,PERCT,U3,U4,PREF,WREF
   WRITE(6,505)
   GO TO 50
525 WRITE(6,504)
   GO TO 50
C          FORMATS
529 FORMAT(2X,13HDESIGN PD,/P4 G15.6)
526 FORMAT(1X,14HDESIGN PD,/P4, G15.6)
527 FORMAT(2X 13HDESIGN ETA-S G15.6)

```

```

528 FORMAT(2X,13HDESIGN ETA-T G15.6)
400 FORMAT(8E10.5)
501 FORMAT(1H1,
1 36X,49HOFF DESIGN PERFORMANCE OF A RADIAL INFLOW TURBINE //
121H TURBINE GEOMETRY AN,G16.6,2X,2HA1,G16.6,2X,2HA3,G16.6,2X,
22FA4,G16.6,2X,2HD1,G16.6 / 19X,2HD3,G16.6,2X,1HL,G17.6,2X,3HAL1,
3G16.6,1X,2HB4,G16.6,2X,1HZ,G16.6 /21H INLET CONDITIONS PO,
4G16.6,2X,2HT0,G16.6 /14H DESIGN SPEEDS,5X,2HU3,G16.6,2X, 2HU4,
5G16.6/20H FLUID PROPERTIES G,G17.6,2X,1HR,G17.6,2X,2HCP,G16.6 ,
62X,3HFPS,G16.6)
401 FORMAT(11)
502 FORMAT( 8HKMODE 1,2X,37HPERFORMANCE RUN VARYING PERCENT SPEED )
503 FORMAT( 8HKMODE 2,2X,31HMANUAL DETERMINATION OF M AND K )
504 FORMAT( 8HKMODE 3,2X,34HAUTOMATIC DETERMINATION OF M AND K )
C
506 FORMAT(12H V/VCR)1,G15.6,9X,7HP2,/PO,,G15.6,14X,2HV3,G15.6,
11X,4HWCR3,G15.6/7X,5HVCR)0,G15.6,9X,7HP2,/P1,,G15.6,11X,5HBETA3,
2G15.6,13X,2HW4,G15.6/8X,5HRHO)0,G15.6,10X,6HALPHA2,G15.6,8X,
38FT,./T.)3,G15.6,11X,4HWCR4,G15.6/12H RHO-VCR)0,G15.6,14X,2HR2,
4G15.6,12X,4HT3,.,G15.6,12X,3HVU4,G15.6/12H PV/PVCR)1,G15.6,
58X,8HVU/VCR)3,G15.6,7X,9HT4,./T3,.,G15.6,7X,8HT4,/T4,.,G15.6 /
612H SMALL-W,G15.6,8X,8HVX/VCR)3,G15.6,9X,7HW/WCR)3,G15.6,6X,
77FP4,/PO,,G15.6 /12H PV/PVCR)0,G15.6,9X,7HV/VCR)3,G15.6,6X,
81CHPVX/PVCR)3,G15.6,11X,4HVCR4,G15.6 )
507 FORMAT(12H V/VCR)0,G15.6,13X,3HVU3,G15.6,9X,7HP4/P4ID,G15.6,
11X,2HV4,G15.6 /12H V/VCR)2,G15.6,14X,2HW3,G15.6,7X,
29HPW/PWCR)4,G15.6,9X,6HP4/P4,,G15.6 /12H PV/PVCR)2,G15.6,13X,
33FVX3,G15.6,9X,7HW/WCR)4,G15.6,9X,6HP4/PO,,G15.6/10X,2HW1,G15.6,
411X,5HETA-E,G15.6,12X,4HW,1D,G15.6,12X,3HT4,,G15.6/9X,3HW1D,G15.6,
511X,5HETA-T,G15.6,11X,5HETA-S,G15.6,12X,3HWEQ,G15.6 /
61F 6X,5HFTA-I,G15.6,14X,2HLE,G15.6,13X,3HDEF,G15.6,13X,
72HLI,G15.6/12H ROTOR-LOSS,G15.6,5X,11HSTATOR-LOSS,G15.6,
812X,3FDER,G15.6,12X,3HDES,G15.6/1H ,6X,5HSPEED,G15.6,8X,
X8FEQ-SPEED,
9G15.6,14X,2HNU,G15.6,7X,8HTHETA-CR,G15.6 )
C
508 FORMAT(7X,5HDELTA,G15.6,7X,9HSPEED-PAR,G15.6,
111X,5HW1-EQ,G15.6,8X,7HPO,/P4,,G15.6/9X,3HNOT,G15.6,12X,4HWTOP,
2G15.6,13X,3FTOP,G15.6,12X,3HDEI,G15.6/
31F 5X,6HPO,/P4,G15.6,13X,3HPHI,G15.6,9X,7HP1,/PO,,G15.6,
410X,5HETA-R,G15.6/1H 8X,3HNSP,G15.6)
END

```

Internal Program Variables and Definitions

| | |
|--------|--|
| AO | A_0 |
| A1 | A_1 |
| A3 | A_3 |
| A4 | A_4 |
| AC | acceleration constant |
| AL1 | α_1 |
| AL2 | α_2 |
| B3 | β_3 |
| B4 | β_4 |
| COSAL1 | $\cos \alpha_1$ |
| COSB3 | $\cos \beta_3$ |
| COSB4 | $\cos \beta_4$ |
| CP | c_p (input variable) |
| D1 | D_1 |
| D3 | D_3 |
| DEE | $\Delta \eta_E$ |
| DEI | $\Delta \eta_I$ |
| DEL | δ |
| DELV | incremental value of $(V/V_{cr})_1$ (input variable) |



| | |
|------------------|-----------------------------|
| DER | $\Delta\eta_R$ |
| DES | $\Delta\eta_S$ |
| DM | Δm |
| ETAD | design value of efficiency |
| ETA _E | η_E |
| ETA _I | η_I |
| ETA _R | η_R |
| ETA _S | η_S |
| ETA _t | η_t |
| EPS | ϵ |
| F | temporary storage |
| F1 | temporary storage |
| F2 | temporary storage |
| F2P | temporary storage |
| F3 | temporary storage |
| F3P | temporary storage |
| FP | temporary storage |
| G | γ |
| G1 | $\gamma + 1$ |
| G2 | $\gamma - 1$ |
| G3 | $(\gamma - 1)/(\gamma + 1)$ |
| H4 | temporary storage |

| | |
|--------|---|
| KMAX | maximum number of k's |
| MODE | if MODE = 1, performance run varying percent speed if MODE = 2, manual determination of m and k if MODE = 3, automatic determination of m and k (input variables) |
| PO | p'_0 |
| POPP4 | p'_0/p_4 |
| POPP4P | p'_0/p'_4 |
| P1PPOP | p'_1/p'_0 |
| P2PPOP | p'_2/p'_0 |
| P2PP1P | p'_2/p'_1 |
| P3P3P | p_3/p'_3 |
| P3P4P | p_3/p'_4 |
| P4 | p_4 |
| P4POP | p_4/p'_0 |
| P4P4ID | $p''_4/p''_{4, id}$ |
| P4P4P | p_4/p'_4 |
| P4PPOP | p'_4/p'_0 |
| P4PP3P | p'_4/p'_3 |
| PDSGN | specified p'_0/p'_4 or p'_0/p_4 (input variable) |
| PERCT | percent of design speed (input variable) |
| PHI | φ |

| | |
|--------|---|
| PIN | $p'_0/144$ |
| PREF | estimate of p'_1/p'_0 to determine k_1 (input variable) |
| PVCRO | $(\rho'V_{cr})_0$ |
| Q | temporary storage |
| Q1 | temporary storage |
| Q2 | temporary storage |
| Q3 | temporary storage |
| Q4 | temporary storage |
| R | R (input variable) |
| R1 | r_1 |
| R2 | r_2 |
| R3 | r_3 |
| RH2VC2 | $(\rho V/\rho'V_{cr})_2$ |
| RHOOP | ρ'_0 |
| RHOVCO | $(\rho V/\rho'V_{cr})_0$ |
| RHWCR4 | $(\rho W/\rho''W_{cr})_4$ |
| RHXVC3 | $(\rho V_x/\rho'V_{cr})_3$ |
| ROTLOS | L_R |
| SINAL1 | $\sin \alpha_1$ |
| SINAL2 | $\sin \alpha_2$ |
| SINB3 | $\sin \beta_3$ |

| | |
|--------|------------------------|
| SPPAR | wN/δ |
| STALOS | L_S |
| TO | T'_0 |
| T4 | T_4 |
| T4P | T'_4 |
| T3PP | T''_3 |
| T4T3 | T''_4/T''_3 |
| T4T4 | $(T'/T'')_4$ |
| THETCR | θ_{cr} |
| TT3 | $(T''/T')_3$ |
| U3 | U_3 (input variable) |
| U31 | initial U_3 |
| U4 | U_4 (input variable) |
| U41 | initial U_4 |
| VO | V_0 |
| V2 | V_2 |
| V3 | V_3 |
| V4 | V_4 |
| VOVCO | $(V/V_{cr})_0$ |
| V1VC1 | see input |



| | |
|--------|---|
| V2VC2 | $(V/V_{cr})_2$ |
| V3VC3 | $(V/V_{cr})_3$ |
| VCRO | $V_{cr,0}$ |
| VCR4 | $V_{cr,4}$ |
| VEND | end value of $(V/V_{cr})_1$ (input variable) |
| VSTART | starting value of $(V/V_{cr})_1$ (input variable) |
| VU3 | $V_{u,3}$ |
| VU4 | $V_{u,4}$ |
| VU3OPT | eq. (B16b) |
| VUVC3 | $(V_u/V_{cr})_3$ |
| VXVC3 | $(V_x/V_{cr})_3$ |
| W3 | W_3 |
| W4 | W_4 |
| W3WC3 | $(W/W_{cr})_3$ |
| WCR3 | $W_{cr,3}$ |
| WEQ | w_{eq} |
| WEQD | see input |
| WID | $\Delta h'_{id,s}$ |
| WIDP | $\Delta h'_{id,t}$ |
| WREF | estimate of W to determine k_1 (input variable) |

| | |
|--------|---------------------|
| WTOP | $w\sqrt{T'_0/p'_0}$ |
| WU3OPT | eq. (B16c) |
| WU4 | $W_{u,4}$ |
| WX3 | $W_{x,3}$ |
| X | temporary storage |
| X1 | temporary storage |
| XJ | constant, 778.16 |
| XK1 | k_1 |
| XLE | L_E |
| XN | N |
| XNOT | N/T'_0 |
| XNSP | NSP |
| XNV | ν |
| XTOP | Γ/p'_{in} |
| ZBR | Z (input variable) |

Subroutine GETK1(XK1:X1)

This routine evaluates the loss coefficient k_1 and a design $(v/v_{cr})_1$ with a specified p'_0/p'_1 and weight flow w where

XK1 loss coefficient k_1

X1 design $(v/v_{cr})_1$

The subroutine is as follows:

```

SUBROUTINE GETKI(XK1,X1)
COMMON /GETK/ P,W,G,A0,A1,AL,T,R,PD
SUB(X,G)=X*(1.-(G-1.)/(G+1.))*X**((1./(G-1.))
VCRO=(64.34*G*R*T/(G+1.))**.5
RFOO=PD/R/T
Q=W/(RHOO*A0*VCRO)
ROCO=Q
X=.1
1 F=SUB(X,G)-Q
F1=(F+Q)/X
FP=-2.*X*X/(G+1.)*F1**(2.-G)+F1
X1=X-F/FP
IF(ABS((X1-X)/X1).LT.1.E-5) GO TO 2
X=X1
GO TO 1
2 VC=X1
Q=W/(P*RHO0*VCRO*A1*COS(AL*.01745))
X=.1
3 F=SUB(X,G)-Q
ROC1=F+Q
F1=(F+Q)/X
FP=-2.*X*X/(G+1.)*F1**(2.-G)+F1
X1=X-F/FP
IF(ABS((X1-X)/X1).LT.1.E-5) GO TO 4
X=X1
GO TO 3
4 XK1=(P**((G-1.)/G)-1.)*(1.-(G-1.)/(G+1.))*X1**2)/
1((G-1.)/(G+1.))*(VO**2+X1**2))*(-1.)
RETURN
END

```

Subroutine GETM(XK, XM, IGO, ETEST, K)

For IGO = 1, this routine computes a value of XM as a function of XK by using equations (B17b) to (B24b). For IGO = 2 this routine varies the value of XK by false positioning until all design specifications are met. The following are defined for this subroutine:

| | |
|-------|---|
| XK | loss coefficient |
| XM | loss coefficient |
| IGO | index for mode of operation |
| ETEST | design value of η_t or η_s |
| K | indicator used in method of false positioning |

The subroutine is as follows:

```

SUBROUTINE GETM(XK,XM,IGD,ETEST,K)
COMMON/GETK/PREF,WREF,G,A0,A1,AL1,TO,K,PO,P4PPOZ,CP,G1,G2,G3,
LEIAT,U3,U4,VU3,B4,T4T3,WCR3,TT3,W3WC3,B3,PHI,P2PF1P,P1PPOP
IF(IGD.GT.1) GO TO 100
P4PPOP=1./P4PPOZ
DIFD=CP*TO*(1.-P4PPOP** (G2/G))
DIF=CHID*ETAT
VL4=(DIF*25033.-VU3*U3)/U4
VL4=-VU4
W4=(VU4-U4)/ABS(SIN(.01745*B4))
W4=-W4
WCR4=WCR3*SQR(T4T3)
W4WC4=W4/WCR4
WU4=W4*SIN(.01745*B4)
T4T4=1.+G3*(U4**2+2.*U4*WU4)/WCR4**2
P4P4ID=P4PPOP/(P2PF1P*P1PPOP*(TT3*T4T3*T4T4)** (G/G2))
T=W3WC3**2/T4T3
T1=-(P4P4IC** (G2/G)-1.)*(1.-G3*W4WC4**2)/G3
XMK=(T1-T*(SIN(B3-PHI))**2)/(W4WC4**2+T)
XM=XMK/XK
RETURN
100 GO TO (101,102),K
101 XK1=XK
E1=ETEST
DIF1=ETEST-ETAT
K=2
XK=XK+.005
RETURN
102 XK2=XK
E2=ETEST
DIF2=ETEST-ETAT
IF(DIF2*DIF1) 106,105,104
104 E1=E2
XK1=XK2
XK=XK+.005
DIF1=DIF2
RETURN
105 XK=XK?
K=6
RETURN
106 XK=-(XK2-XK1)/(DIF2-DIF1)*DIF1+XK1
K=6
RETURN
END

```

5
6
7
8
9
10
11

Lewis Research Center,
National Aeronautics and Space Administration,
Cleveland, Ohio, December 6, 1968,
125-23-02-10-22.

APPENDIX A

SYMBOLS

| | |
|--------------------|---|
| A | area, sq ft |
| c_p | specific heat at constant pressure, Btu/(lb)($^{\circ}$ R) |
| D | diameter, ft |
| g | dimensional constant, 32.1741 ft/sec ² |
| $\Delta h'$ | specific turbine work, Btu/lb |
| $\Delta h'_{id,s}$ | ideal turbine work based on inlet total to exit static pressure ratio, Btu/lb |
| $\Delta h'_{id,t}$ | ideal turbine work based on inlet total to exit total pressure ratio, Btu/lb |
| J | mechanical equivalent of heat, 778.029 ft-lb/Btu |
| k_1, k, m | loss coefficients, dimensionless |
| L | kinetic energy loss, Btu/lb |
| l | axial distance, ft |
| N | turbine speed, rpm |
| NSP | specific speed |
| p | pressure, lb/sq ft |
| p_{in} | turbine inlet pressure, psia |
| R | gas constant, (ft-lb)/(lb)($^{\circ}$ R) |
| r | radius, ft |
| T | absolute temperature, $^{\circ}$ R |
| U | blade speed, ft/sec |
| V | absolute velocity of gas, ft/sec |
| W | gas velocity relative to rotor, ft/sec |
| w | mass flow rate, lb/sec |
| Z | number of rotor blades at tip of rotor |
| α | absolute gas angle, angle between absolute velocity vector and meridional plane at mean channel, positive when direction of tangential velocity component and direction of wheel velocity agree and negative when they disagree |

| | |
|--------------------------|--|
| β | relative gas angle, angle between the velocity vector relative to wheel and meridional plane at mean channel; same sign convention applies as for α |
| Γ | torque, ft-lb |
| γ | ratio of specific heat at constant pressure to specific heat at constant volume |
| δ | ratio of turbine inlet total pressure to U.S. standard atmospheric pressure, $p'_0/2116.22$ |
| ϵ | function of γ used in relating parameters to those using air inlet conditions at U.S. standard sea-level conditions |
| | $\frac{\gamma^*}{\gamma} \left[\frac{\left(\frac{\gamma + 1}{2} \right)^{\gamma/(\gamma-1)}}{\left(\frac{\gamma^* + 1}{2} \right)^{\gamma^*/(\gamma^*-1)}} \right]$ |
| η_E, η_I, η_R | modified efficiency figures used in studying loss distribution in turbine, dimensionless |
| η_S | efficiency based on ratio of inlet total to exit static pressure |
| η_t | efficiency based on ratio of inlet total to exit total pressure |
| θ_{cr} | squared ratio of critical velocity at turbine inlet to critical velocity at U.S. standard atmospheric temperature, $\left(V_{cr, 0} / 1019.46 \right)^2$ |
| ν | blade-jet speed ratio, $U_3 / \sqrt{2gJ \Delta h'_{id, s}}$ |
| φ | optimum rotor incidence angle, deg |
| ρ | gas density, lb/cu ft |
| Subscripts: | |
| cr | condition corresponding to Mach 1 |
| E | exit |
| eq | air equivalent (U.S. standard sea level) |
| I | incidence |
| id | ideal |
| opt | optimum |
| R | rotor |
| S | stator |
| u | tangential component, positive when its direction agrees with that of wheel velocity and negative when it disagrees |

- x meridional component, component in plane containing axis of rotation
- 0 station at turbine inlet
- 1 station immediately upstream of stator exit
- 2 station where flow from stator exit is assumed to occupy entire cylindrical area
- 3 station immediately upstream of rotor inlet
- 4 station immediately downstream of rotor exit

Superscripts:

- (') absolute total state
- ('') total state relative to rotor
- (*) U.S. standard sea-level conditions (temperature, 518.67⁰ R; pressure, 14.696 psia)

APPENDIX B

THERMODYNAMIC EQUATIONS

The equations in this appendix are listed in the order used by the computer. A complete set of calculations is made for each specified value of $(V/V_{cr})_1$ at each speed.

$$\left(\frac{\rho V}{\rho' V_{cr}}\right)_1 = \left(\frac{V}{V_{cr}}\right)_1 \left[1 - \frac{\gamma - 1}{\gamma + 1} \left(\frac{V}{V_{cr}}\right)_1^2 \right]^{1/(\gamma-1)} \quad (B1)$$

Equations (B2) to (B5) are to be solved simultaneously for p'_1/p'_0 :

$$w = \frac{p'_1}{p'_0} \left(\frac{\rho V}{\rho' V_{cr}}\right)_1 (p' V_{cr})_0 A_1 \cos \alpha_1 \quad (B2)$$

$$\left(\frac{\rho V}{\rho' V_{cr}}\right)_0 = \frac{w}{(\rho' V_{cr} A)_0} \quad (B3)$$

Calculate $(V/V_{cr})_0$ from

$$\left(\frac{\rho V}{\rho' V_{cr}}\right)_0 = \left(\frac{V}{V_{cr}}\right)_0 \left[1 - \frac{\gamma - 1}{\gamma + 1} \left(\frac{V}{V_{cr}}\right)_0^2 \right]^{1/(\gamma-1)} \quad (B4)$$

$$\frac{p'_1}{p'_0} = \left\{ 1 - \frac{\frac{\gamma - 1}{\gamma + 1} k_1 \left[\left(\frac{V}{V_{cr}}\right)_0^2 + \left(\frac{V}{V_{cr}}\right)_1^2 \right]}{1 - \frac{\gamma - 1}{\gamma + 1} \left(\frac{V}{V_{cr}}\right)_1^2} \right\}^{\gamma/(\gamma-1)} \quad (B5)$$

$$\left(\frac{V}{V_{cr}}\right)_2 = \left[\frac{k_1 + 1}{k + 1} \left(\frac{V}{V_{cr}}\right)_1^2 - \frac{k - k_1}{k + 1} \left(\frac{V}{V_{cr}}\right)_0^2 \right]^{1/2} \quad (\text{B6a})$$

$$\left(\frac{\rho V}{\rho' V_{cr}}\right)_2 = \left(\frac{V}{V_{cr}}\right)_2 \left[1 - \frac{\gamma - 1}{\gamma + 1} \left(\frac{V}{V_{cr}}\right)_2^2 \right]^{1/(\gamma-1)} \quad (\text{B6b})$$

$$\frac{p'_2}{p'_0} = \left\{ 1 - \frac{\frac{\gamma - 1}{\gamma + 1} k \left[\left(\frac{V}{V_{cr}}\right)_0^2 + \left(\frac{V}{V_{cr}}\right)_2^2 \right]}{1 - \frac{\gamma - 1}{\gamma + 1} \left(\frac{V}{V_{cr}}\right)_2^2} \right\}^{\gamma/(\gamma-1)} \quad (\text{B7})$$

$$\frac{p'_2}{p'_1} = \frac{\frac{p'_2}{p'_0}}{\frac{p'_1}{p'_0}} \quad (\text{B8})$$

$$\alpha_2 = \tan^{-1} \left[\frac{\pi D_1 l \left(\frac{\rho V}{\rho' V_{cr}}\right)_2 \left(\frac{p'_2}{p'_1}\right) \sin \alpha_1}{\left(\frac{\rho V}{\rho' V_{cr}}\right)_1 A_1 \cos \alpha_1} \right] \quad (\text{B9})$$

$$\gamma_2 = \gamma_1 \frac{\sin \alpha_1}{\sin \alpha_2} \quad (\text{B10})$$

$$\left(\frac{V_u}{V_{cr}}\right)_3 = \frac{\gamma_2}{\gamma_3} \left(\frac{V}{V_{cr}}\right)_2 \sin \alpha_2 \quad (B11)$$

Determine $\left(\frac{V_x}{V_{cr}}\right)_3$ from

$$\left(\frac{V_x}{V_{cr}}\right)_3 = \frac{\frac{\gamma_2}{\gamma_3} \left(\frac{\rho V}{\rho' V_{cr}}\right)_2 \cos \alpha_2}{\left\{1 - \frac{\gamma - 1}{\gamma + 1} \left[\left(\frac{V_x}{V_{cr}}\right)_3^2 + \left(\frac{V_u}{V_{cr}}\right)_3^2 \right] \right\}^{1/(\gamma-1)}} \quad (B12)$$

$$\left(\frac{V}{V_{cr}}\right)_3 = \left[\left(\frac{V_x}{V_{cr}}\right)_3^2 + \left(\frac{V_u}{V_{cr}}\right)_3^2 \right]^{1/2} \quad (B13a)$$

$$\left(\frac{\rho V}{\rho' V_{cr}}\right)_3 = \left(\frac{V}{V_{cr}}\right)_3 \left[1 - \frac{\gamma - 1}{\gamma + 1} \left(\frac{V}{V_{cr}}\right)_3^2 \right]^{1/(\gamma-1)} \quad (B13b)$$

$$V_{u,3} = V_{cr,0} \left(\frac{V_u}{V_{cr}}\right)_3 \quad (B14a)$$

$$V_{x,3} = V_{cr,0} \left(\frac{V_x}{V_{cr}}\right)_3 \quad (B14b)$$

$$V_3 = V_{cr,0} \left(\frac{V}{V_{cr}}\right)_3 \quad (B14c)$$

$$W_3 = \left(V_3^2 - 2U_3 V_{u,3} + U_3^2 \right)^{1/2} \quad (B14d)$$

$$\cos \beta_3 = \frac{V_{x,3}}{W_3} \quad (\text{B14e})$$

$$\beta_3 = \sin^{-1} \frac{V_{u,3} - U_3}{W_3} \quad (\text{B14f})$$

$$\left(\frac{T''}{T'}\right)_3 = 1 - \frac{\gamma - 1}{\gamma + 1} \left(\frac{2U_3 V_{u,3} - U_3^2}{V_{cr,0}^2} \right) \quad (\text{B15a})$$

$$T''_3 = T'_0 \left(\frac{T''}{T'}\right)_3 \quad (\text{B15b})$$

$$\frac{T''_4}{T''_3} = 1 - \frac{U_3^2 - U_4^2}{2gJc_p T''_3} \quad (\text{B15c})$$

$$\left(\frac{W}{W_{cr}}\right)_3 = \frac{\left(\frac{V_x}{V_{cr}}\right)_3}{\left(\frac{T''}{T'}\right)_3^{1/2} \cos \beta_3} \quad (\text{B15d})$$

$$\left(\frac{\rho V_x}{\rho' V_{cr}}\right)_3 = \left(\frac{\rho V}{\rho' V_{cr}}\right)_3 \frac{\left(\frac{V_x}{V_{cr}}\right)_3}{\left(\frac{V}{V_{cr}}\right)_3} \quad (\text{B15e})$$

Calculate the angle φ as follows:

$$\frac{V_{u,3,opt}}{U_3} = 1 - \frac{1.98}{Z} \quad (\text{B16a})$$

$$V_{u, 3, \text{opt}} = U_3 \left(\frac{V_{u, 3, \text{opt}}}{U_3} \right) \quad (\text{B16b})$$

$$W_{u, 3, \text{opt}} = V_{u, 3, \text{opt}} - U_3 \quad (\text{B16c})$$

$$\varphi = \tan^{-1} \frac{W_{u, 3, \text{opt}}}{V_{x, 3}} \quad (\text{B16d})$$

The following three equations are to be solved simultaneously for $(W/W_{\text{cr}})_4$ and $p''_4/p''_{4, \text{id}}$:

$$\left(\frac{\rho W}{\rho'' W_{\text{cr}}} \right)_4 = \frac{\left(\frac{\rho V_x}{\rho' V_{\text{cr}}} \right)_3 \frac{A_3}{A_4 \cos \beta_4}}{\frac{p''_4}{p''_{4, \text{id}}} \left(\frac{T''}{T'} \right)_3^{(\gamma+1)/2(\gamma-1)} \left(\frac{T''_4}{T''_3} \right)^{(\gamma+1)/2(\gamma-1)}} \quad (\text{B17a})$$

$$\frac{p''_4}{p''_{4, \text{id}}} = 1 - \frac{\left\{ \frac{\gamma-1}{\gamma+1} \left[\text{mk} \left(\frac{W}{W_{\text{cr}}} \right)_4^2 + \frac{\left(\frac{W}{W_{\text{cr}}} \right)_3^2}{\frac{T''_4}{T''_3}} \left[\text{mk} + \sin^2 (\beta_3 - \varphi) \right] \right\}^{\gamma/(\gamma-1)}}{1 - \frac{\gamma-1}{\gamma+1} \left(\frac{W}{W_{\text{cr}}} \right)_4^2} \quad (\text{B17b})$$

$$\left(\frac{\rho W}{\rho'' W_{\text{cr}}} \right)_4 = \left(\frac{W}{W_{\text{cr}}} \right)_4 \left[1 - \frac{\gamma-1}{\gamma+1} \left(\frac{W}{W_{\text{cr}}} \right)_4^2 \right]^{1/(\gamma-1)} \quad (\text{B17c})$$

$$W_{\text{cr}, 3} = \frac{W_3}{\left(\frac{W}{W_{\text{cr}}}\right)_3} \quad (\text{B18a})$$

$$W_{\text{cr}, 4} = W_{\text{cr}, 3} \left(\frac{T''_4}{T''_3}\right)^{1/2} \quad (\text{B18b})$$

$$W_4 = W_{\text{cr}, 4} \left(\frac{W}{W_{\text{cr}}}\right)_4 \quad (\text{B18c})$$

$$\left(\frac{T'}{T''}\right)_4 = 1 + \frac{\gamma - 1}{\gamma + 1} \left(\frac{U_4^2 + 2U_4 W_{u, 4}}{W_{\text{cr}, 4}^2}\right) \quad (\text{B18d})$$

$$W_{u, 4} = W_4 \sin \beta_4 \quad (\beta_4 \text{ will be negative}) \quad (\text{B18e})$$

$$\frac{p'_4}{p'_0} = \frac{p'_2}{p'_1} \frac{p'_1}{p'_0} \frac{p''_4}{p''_{4, \text{id}}} \left[\left(\frac{T'''}{T'''}\right)_3 \left(\frac{T''_4}{T''_3}\right) \left(\frac{T'}{T''}\right)_4 \right]^{\gamma/(\gamma-1)} \quad (\text{B19a})$$

$$V_{\text{cr}, 4} = W_{\text{cr}, 4} \left(\frac{T'}{T''}\right)_4^{1/2} \quad (\text{B19b})$$

$$W_4 = W_{\text{cr}, 4} \left(\frac{W}{W_{\text{cr}}}\right)_4 \quad (\text{B19c})$$

$$V_4 = \left[W_4^2 \cos^2 \beta_4 + (U_4 + W_4 \sin \beta_4)^2 \right]^{1/2} \quad (\text{B19d})$$

$$\frac{p_4}{p'_4} = \left[1 - \frac{\gamma - 1}{\gamma + 1} \left(\frac{V}{V_{\text{cr}}}\right)_4^2 \right]^{\gamma/(\gamma-1)} \quad (\text{B20a})$$

$$\frac{p_4}{p_0} = \frac{p_4'}{p_0'} \frac{p_4}{p_4'} \quad (\text{B20b})$$

$$V_{u,4} = U_4 + W_4 \sin \beta_4 \quad (\text{B21a})$$

$$T_4' = \frac{T_4'}{T_4''} \frac{T_4''}{T_3'} T_3' \quad (\text{B21b})$$

$$\Delta h' = \frac{V_{u,3} U_3 - V_{u,4} U_4}{gJ} \quad (\text{B22})$$

$$\Delta h'_{id,t} = c_p T_0' \left[1 - \left(\frac{p_4'}{p_0'} \right)^{(\gamma-1)/\gamma} \right] \quad (\text{B23a})$$

$$\Delta h'_{id,s} = c_p T_0' \left[1 - \left(\frac{p_4}{p_0'} \right)^{(\gamma-1)/\gamma} \right] \quad (\text{B23b})$$

$$\eta_t = \frac{\Delta h'}{\Delta h'_{id,t}} \quad (\text{B24a})$$

$$\eta_s = \frac{\Delta h'}{\Delta h'_{id,s}} \quad (\text{B24b})$$

The foregoing equations have given the flow conditions through the turbine and the efficiencies. The following equations give quantities which are useful for special purposes, such as identifying sources of loss in the turbine:

$$L_R = mk \frac{W_3^2 + W_4^2}{2gJ} \quad (\text{rotor loss}) \quad (\text{B25})$$

$$V_0 = V_{cr, 0} \left(\frac{V}{V_{cr}} \right)_0 \quad (\text{B26a})$$

$$V_2 = V_{cr, 0} \left(\frac{V}{V_{cr}} \right)_2 \quad (\text{B26b})$$

$$L_S = k \frac{V_0^2 + V_2^2}{2gJ} \quad (\text{stator loss}) \quad (\text{B26c})$$

$$\Delta \eta_R = \frac{L_R}{\Delta h'_{id, s}} \quad (\text{fractional rotor loss}) \quad (\text{B27})$$

$$\Delta \eta_S = \frac{L_S}{\Delta h'_{id, s}} \quad (\text{fractional stator loss}) \quad (\text{B28})$$

$$L_E = \frac{V_4^2}{2gJ} \quad (\text{exit loss}) \quad (\text{B29a})$$

$$\Delta \eta_E = \frac{L_E}{\Delta h'_{id, s}} \quad (\text{fractional exit loss}) \quad (\text{B29b})$$

$$L_I = \frac{W_3^2 \sin^2(\beta_3 - \varphi)}{2gJ} \quad (\text{incidence loss}) \quad (\text{B30a})$$

$$\Delta \eta_I = \frac{L_I}{\Delta h'_{id, s}} \quad (\text{fractional incidence loss}) \quad (\text{B30b})$$

$$\left. \begin{aligned} \eta_E &= \eta_S + \Delta\eta_E \\ \eta_R &= \eta_E + \Delta\eta_R \\ \eta_I &= \eta_R + \Delta\eta_I \end{aligned} \right\} \quad (\text{B31})$$

$$\theta_{\text{cr}} = \left(\frac{V_{\text{cr}, 0}}{1019.46} \right)^2 \quad (\text{B32a})$$

$$\epsilon = \frac{1.40}{\gamma} \left[\frac{\left(\frac{\gamma + 1}{2} \right)^{\gamma/(\gamma-1)}}{1.8929} \right] \quad (\text{B32b})$$

$$\delta = \frac{p'_0}{2116.22} \quad (\text{B32c})$$

$$w^{\text{eq}} = w \frac{\sqrt{\theta_{\text{cr}}}}{\delta} \epsilon \quad (\text{equivalent weight flow}) \quad (\text{B33})$$

$$\frac{wN}{\delta} \epsilon \quad (\text{weight flow-speed parameter}) \quad (\text{B34})$$

$$N_{\text{eq}} = \frac{N}{\sqrt{\theta_{\text{cr}}}} \quad (\text{equivalent speed}) \quad (\text{B35})$$

$$\nu = \frac{U_3}{\sqrt{2gJ \Delta h'_{\text{id}, s}}} \quad (\text{blade-jet speed ratio}) \quad (\text{B36})$$

$$\Delta h'_{\text{eq}} = \frac{\Delta h'}{\theta_{\text{cr}}} \quad (\text{equivalent specific work}) \quad (\text{B37})$$

$$\text{NOT} = \frac{N}{\sqrt{T_0}} \quad (\text{B38})$$

$$W_{TOP} = \frac{w \sqrt{T'_0}}{p'_{in}} \quad (B39)$$

where $p'_{in} = p'_0/144$

$$TOP = \frac{\Gamma}{p'_{in}} \quad (B40)$$

where $\Gamma = 89\ 155\ w\ \Delta h'/N$ (in. -lb)

$$NSP = \frac{N\sqrt{Q}}{H^{3/4}} \quad (\text{specific speed}) \quad (B41)$$

where $Q = w/\rho_4$ and $H = J\ \Delta h'_{id,t}$.

For calculation of ρ_4 ,

$$p_4 = \frac{p_4}{p'_0} p'_0$$

$$T_4 = T'_4 \left[1 - \frac{\gamma - 1}{\gamma + 1} \left(\frac{V}{V_{cr4}} \right)^2 \right]$$

$$\rho_4 = \frac{p_4}{RT_4}$$

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