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(Engineering Symbols)

| Symbol | FøRTRAN ${ }^{1}$ <br> name | Description |
| :---: | :---: | :---: |
| A | A | cross-sectional area of local section, $\mathrm{m}^{2}\left(\mathrm{ft}^{2}\right)$ |
| $A_{F}$ |  | total cross-sectional flow area at drive fan(s), $m^{2}\left(f t^{2}\right)$ |
| AFLOW | AL | cross-sectional area of local flow, $\mathrm{m}^{2}$ (ft ${ }^{2}$ ) |
| $\mathrm{A}_{0}$ | A0 | cross-sectional flow area of test section at upstream end, $m^{2}\left(f t^{2}\right)$ |
| $\mathrm{A}_{1}$ | A1 | cross-sectional flow area of section at upstream end, $m^{2}\left(f t^{2}\right)$ |
| $\mathrm{A}_{2}$ | A2 | cross-sectional flow area of section at is downstream end, $m^{2}\left(f t^{2}\right)$ |
| $A_{*}$ | ASTAR | cross-sectional area for sonic flow at specified flow conditions, $\mathrm{m}^{2}\left(\mathrm{ft}^{2}\right)$ |
| AR | AR | cross-sectional flow area ratio of upstream and downstream ends of section |
| $\mathrm{a}_{T}$ | AT | speed of sound in still gas, computed at total (stagnation) conditions, $\mathrm{m} / \mathrm{sec}$ (ft/sec) |
| $a_{0}$ | ASO | speed of sound in moving flow at upstream end of test section, $\mathrm{m} / \mathrm{sec}$ ( $\mathrm{ft} / \mathrm{sec}$ ) |
| B |  | dummy constraint used in defining the friction term of turning vane loss function |
| $C_{\text {D }}$ | CD | ```drag coefficient of flow obstructions: drag/qS``` |
| $c_{v}$ | CHøRD | chord of turning vanes, $m$ ( $f t$ ) |
| D | D | ```cross-sectional diameter of circular duct, m (ft)``` |
| $\mathrm{De}_{1}$ |  | cross-section diameter at the upstream end of an equivalent circular duct with equal area, m (ft) |

[^1]| Symbol | FøRTRAN <br> name | Description |
| :---: | :---: | :---: |
| $\mathrm{D}_{2}$ |  | cross-section diameter at the downstream end of an equivalent circular duct with equal area, m (ft) |
| $\mathrm{D}_{\mathrm{h}}$ | DH | $\begin{aligned} & \text { hydraulic diameter: } \frac{4 \times(\text { cross-sectional area })}{\text { perimeter }}, \\ & m(f t) \end{aligned}$ |
| ER | ER | energy ratio: ratio of energy of flow at the test section to the output energy of the fans |
| $f(\emptyset)$ | $\begin{aligned} & \text { FKTV1 } \\ & \text { FKTV2 } \end{aligned}$ | function defining turning vane loss parameter $\mathrm{K}_{\mathrm{TV}}$ |
| K | EK | local total pressure loss coefficient of section: $\frac{\Delta p_{T}}{q}$ |
| K CONTRACTION | EKCNTR | local total pressure loss coefficient from contracting portion of thick-airfoil flow straighteners |
| KDIFFUSION | EKD | local total pressure loss coefficient from diffusing portion of multi-loss-type sections |
| $\mathrm{K}_{\text {EXP }}$ | EKEXP | net expansion loss coefficient for diffusers |
| $\mathrm{K}_{\text {EXP }}{ }_{\text {Additional }}$ | EKADD | additional diffuser expansion loss coefficient due only to more diffusion in one plane than the other |
| $\mathrm{K}_{\text {EXP }}{ }_{\text {Basic }}$ | EKBASE | basic diffuser expansion loss factor coefficient for three-dimensional diffusion |
| $\mathrm{K}_{\text {EXP }}{ }_{\text {Circular }}$ | EKC | expansion loss coefficient for conical diffusers |
| $\mathrm{K}_{\text {EXP }}{ }_{\text {Rectangular }}$ | EK2DR | expansion loss coefficient for a twodimensional, rectangular cross-section diffuser |
| $\mathrm{K}_{\text {EXP }}{ }_{\text {Square }}$ | EKS | expansion loss coefficient for threedimensional expansion in square cross-section diffusers |
| $\mathrm{K}_{\text {EXP }_{2 \mathrm{D}_{\text {Average }}}}$ | EK2DCS | estimated expansion loss value for a twodimensional diffuser (one with expansion in only one plane) with cross-section shape of some square/circular hybrid |

Symbol
$\mathrm{KEXP}_{\text {2D }}$ Circular
$\mathrm{K}_{\mathrm{EXP}}^{3 \mathrm{D}_{\text {Average }}}{ }$

Kexpansion

## Krriction

K FRICTION (CONICAL)
$\mathrm{K}_{\text {MESH }}$
$K_{\text {Ref. }} 9$
$K_{\text {RN }}$
KROTATION
$\mathrm{K}_{\mathrm{TV}}$
$\mathrm{K}_{\mathrm{TV}}{ }_{90}$
$K_{v}$
$K_{\text {VANED DIFFUSER }}$
$\mathrm{K}_{\mathrm{o}}$

EK2DC

EKCSAV

EKMESH

EKTV
EKTV90

EKV

EKO
estimated expansion loss value for a hypothetical two-dimensional diffuser with circular sides:
$\mathrm{K}_{\text {EXP }}{ }_{2 D_{\text {Rectangular }}}\left(\frac{\mathrm{K}_{\text {EXP }}{ }_{\text {Circular }}}{\mathrm{K}_{\text {EXP }} \text { Square }}\right)$
estimated expansion loss coefficient for threedimensional, combination circular and square cross-section diffuser
diffuser loss coefficient due to expansion:
$K_{\operatorname{EXP}}\left(\frac{\mathrm{AR}-1}{\mathrm{AR}}\right)^{2}$
turning vane loss due to friction
diffuser loss due to friction for the equivalent conical diffuser
mesh screen-type loss parameter
diffuser loss factor presented in reference 9: $\Delta p / q$
$\overline{[(A R-1) / A R]^{2}}$
mesh screen Reynolds number sensitivity factor turning vane loss coefficient due to rotation
turning vane loss coefficient
turning vane loss parameter for given vanes at a $90^{\circ}$ turn
local total pressure loss coefficient for vaned diffusers
local total pressure loss coefficient for vaned diffuser, $\mathrm{K}_{\mathrm{v}}\left(\frac{\mathrm{AR}-1}{\mathrm{AR}}\right)^{2}$
section total pressure loss coefficient referred to test section conditions: $\frac{\Delta \mathrm{p}_{\mathrm{T}}}{\mathrm{q}_{\mathrm{o}}}$

| Symbol | $\begin{array}{c}\text { FøRTRAN } \\ \text { name }\end{array}$ | Description |
| :---: | :---: | :---: |
| $\mathrm{K}_{\text {O }_{\text {DRAG }}}$ |  | flow obstruction (drag item) total pressure loss coefficient referred to test section conditions |
| L | EL | centerline length of section, m (ft) |
| $\ell$ |  | characteristic dimension on which Reynolds number is based |
| M | AMACH | local Mach number |
| $\mathrm{M}_{0}$ | EMO | Mach number at upstream end of test section |
| N | N | section assigned sequence number for order of occurrence in circuit |
| ${ }^{\text {P DRAG }}$ |  | power loss due to drag of flow obstruction, W (hp) |
| $\mathrm{P}_{\text {INPUT }}$ | PWRIP | tunnel drive power required to be input to flow by the fans, W (hp) |
| $\mathrm{P}_{\text {INPUT }}{ }_{\text {DRAG }}$ |  | power input required to overcome drag of flow obstruction, W (hp) |
| $\mathrm{P}_{\text {REQUIRED }}$ | PWR $\varnothing$ P | total fan motor output power required to drive wind tunnel at specified speed, W (hp) |
| P |  | local static pressure, $\mathrm{N} / \mathrm{m}^{2}\left(1 \mathrm{~b} / \mathrm{ft}^{2}\right)$ |
| ${ }^{p}$ T | PT | ```tunnel total (stagnation) pressure, N/m (1b/ft }\mp@subsup{}{}{2}\mathrm{ )``` |
| ${ }^{P} \mathrm{~T}_{\text {ATM }}$ | PATM | $\begin{aligned} & \text { atmospheric (barometric) pressure, } \mathrm{N} / \mathrm{m}^{2} \\ & \left(\mathrm{lb} / \mathrm{ft}^{2}\right) \end{aligned}$ |
| ${ }^{\mathrm{P}} \mathrm{T}_{\text {SC }}$ |  | total (stagnation) pressure in the circuit settling chamber, $\mathrm{N} / \mathrm{m}^{2}\left(\mathrm{lb} / \mathrm{ft}^{2}\right)$ |
| q |  | local dynamic pressure: $\frac{\rho V^{2}}{2}, \mathrm{~N} / \mathrm{m}^{2}\left(\mathrm{lb} / \mathrm{ft}^{2}\right)$ |
| $\mathrm{q}_{0}$ | Q0 | test section dynamic pressure: $\frac{\rho_{\mathrm{O}} \mathrm{V}_{\mathrm{O}}{ }^{2}}{2}, \mathrm{~N} / \mathrm{m}^{2}$ (lb/ft ${ }^{2}$ ) |
| R | R | gas constant, $\mathrm{m}^{2} / \mathrm{sec}^{2}{ }^{\circ} \mathrm{K}\left(\mathrm{ft}{ }^{2} / \mathrm{sec}^{2}{ }^{\circ} \mathrm{R}\right)$ |
| $\mathrm{R}_{\mathrm{e}}$ |  | equivalent radius: $\sqrt{\mathrm{A} / \pi}, \mathrm{m}$ (ft) |


| Symbol | FøRTRAN <br> name | Description |
| :---: | :---: | :---: |
| RN | RN | Reyno1ds number: $\frac{\rho V \ell}{\mu}$ |
| $\mathrm{RN}_{\text {REF }}$ | RNREF | reference Reynolds number at which turning vane $90^{\circ}$ loss parameter, $\mathrm{K}_{\mathrm{TV}_{90}}$, was determined |
| S |  | drag area of flow obstruction (i.e., area for which $C_{D}$ is determined), $\mathrm{m}^{2}\left(\mathrm{ft}^{2}\right)$. |
| $s$ |  | distance along diffuser wall, m (ft) |
| $s_{2}$ |  | length of diffuser, taken along wall, m (ft) |
| T |  | tunnel temperature in moving flow, ${ }^{\circ} \mathrm{K}\left({ }^{\circ} \mathrm{R}\right)$ |
| $\mathrm{T}_{\mathrm{T}}$ | TT | tunnel total (stagnation) temperature, ${ }^{\circ} \mathrm{K}\left({ }^{\circ} \mathrm{R}\right)$ |
| V | V | local flow velocity, m/sec (ft/sec) $\quad \vdots$ |
| $\mathrm{V}_{\mathrm{F}}$ |  | flow velocity at the drive fan(s), $\mathrm{m} / \mathrm{sec}$ (ft/sec) |
| $\mathrm{v}_{\text {SYSTEM }}$ |  | ```flow velocity in a multiple-duct section, m/sec (ft/sec)``` |
| $\mathrm{V}_{\mathrm{o}}$ | vo | ```test section upstream-end flow velocity, m/sec (ft/sec)``` |
| X |  | location of inflection point in contraction wall (distance from upstream end), m (ft) |
| $\gamma$ | G | specific heat ratio of gas |
| $\Delta$ | RUFNES | surface roughness in honeycomb cells, m (ft) |
| $\triangle E R$ |  | difference between estimated and true circuit energy ratios; i.e., error in energy ratio estimate |
| $\Delta \mathrm{P}_{\mathrm{F}}$ |  | static pressure rise across the fan(s), $N / m^{2}$ ( $1 \dot{\mathrm{~b}} / \mathrm{ft}^{2}$ ) |
| $\Delta \mathrm{P}_{\mathrm{T}}$ |  | $\begin{aligned} & \text { total pressure drop through a section, } \mathrm{N} / \mathrm{m}^{2} \\ & \left(1 \mathrm{~b} / \mathrm{ft}^{2}\right) \end{aligned}$ |
| $\Delta \mathrm{p}_{\text {TDUCT }}$ |  | total pressure drop through a single duct of a multiple-duct section, $\mathrm{N} / \mathrm{m}^{2}\left(\mathrm{lb} / \mathrm{ft}^{2}\right)$ |
| ${ }^{\Delta \mathrm{P}_{\mathrm{T}}}{ }_{\mathrm{F}}$ |  | total pressure rise across the $f\left(\mathrm{fan}(\mathrm{s}), \mathrm{N} / \mathrm{m}^{2}\right.$ ( $\mathrm{lb} / \mathrm{ft}^{2}$ ) |

Symbol
$\Delta \mathrm{p}_{\mathrm{T}_{\text {DUCT }}}$
$\Delta \mathrm{p}_{\mathrm{T}_{\text {SYSTEM }}}$
$\Delta \mathrm{p}_{\mathrm{T}_{\text {TOTAL }}}$
$\Delta \mathrm{p}_{\mathrm{W}_{\mathrm{i}}}$
$\Delta \mathrm{P}_{\text {REQUIRED }}$
$\Delta V_{0}$
$\Delta \varepsilon$
$\delta_{s}$
$\varepsilon$
$\eta_{E}$

| $\eta_{F}$ | ETAFAN |
| :--- | :--- |
| $\theta$ | TH |
| $\lambda$ | SLAMDA |
| $\mu$ | EMU |
| $\mu_{\text {std }}$ | EMUSTD |

${ }^{\mu} T$
$\qquad$
average total pressure rise across a single fan, $\mathrm{N} / \mathrm{m}^{2}\left(\mathrm{lb} / \mathrm{ft}^{2}\right)$
total pressure drop through a multiple-duct section, $N / m^{2}\left(\mathrm{lb} / \mathrm{ft}^{2}\right)$
summation of all total pressure drops through the wind tunnel circuit, $N / \mathrm{m}^{2}\left(1 \mathrm{~b} / \mathrm{ft}^{2}\right)$
local pressure difference across wind tunnel wall, $N / \mathrm{m}^{2}\left(1 \mathrm{~b} / \mathrm{ft}^{2}\right)$
difference between true and estimated required drive power levels for given levels of operating velocity and fan efficiency; i.e., error in required power estimate, $W$ (hp)
difference between estimated and true test section operating velocity for given power and fan efficiency levels; i.e., error in operating velocity estimate, $\mathrm{m} / \mathrm{sec}(\mathrm{ft} / \mathrm{sec}$ )
increment of flow-obstruction downstream influence factor greater than unity: $\varepsilon-1$ (greater than or equal to zero)
diffuser side length ratio: ratio of change in height to change in width from upstream to downstream end, or its inverse, whichever is less than or equal to unity
flow-obstruction downstream influence coefficient (greater than or equal to unity)
drive motor electrical efficiency, percent
fan aerodynamic efficiency, percent
diffuser half-angle, rad
friction coefficient for smooth pipes flow viscosity, $N \mathrm{sec} / \mathrm{m}^{2}\left(\mathrm{lb} \mathrm{sec} / \mathrm{ft}^{2}\right)$
standard-day value of viscosity, $\mathrm{N} \sec / \mathrm{m}^{2}$ (lb sec/ft ${ }^{2}$ )
reference viscosity at a known temperature, computed for still gas (stagnation conditions), $\mathrm{N} \sec / \mathrm{m}^{2}\left(\mathrm{lb} \mathrm{sec} / \mathrm{ft}^{2}\right)$

| Symbol | FgRTRAN name | Description |
| :---: | :---: | :---: |
| $v$ | ENU | kinetic viscosity of gas, $\mathrm{m}^{2} / \mathrm{sec}\left(\mathrm{ft}^{2} / \mathrm{sec}\right)$ |
| $\rho$ | RHøS | local static density, $\mathrm{N} \mathrm{sec}^{2} / \mathrm{m}^{4}$ ( $1 \mathrm{~b} \mathrm{sec}^{2} / \mathrm{ft}^{4}$ ) |
| $\rho_{\mathrm{F}}$ | RHØSF | static density at the fan(s), $N \sec ^{2} / \mathrm{m}^{4}$ (1b $\sec ^{2} / f t^{4}$ ) |
| $\rho_{\mathrm{T}}$ | RHめT | density computed for total (stagnation) conditions, $N \sec ^{2} / \mathrm{m}^{4}\left(1 \mathrm{~b} \mathrm{sec}{ }^{2} / \mathrm{ft}^{4}\right)$ |
| $\rho_{0}$ | RHøSO | static density at upstream end of test section, $\mathrm{N} \sec ^{2} / \mathrm{m}^{4}\left(\mathrm{lb} \sec ^{2} / \mathrm{ft}^{4}\right)$ |
| $\sum_{i=1}^{N} k_{o_{i}}$ | SUMEKO | summation of section total pressure losses referenced to test section conditions |
| $\sum_{i=1}^{N} L_{i}$ | SUMEL | summation of section centerline lengths, m (ft) |
| $\phi$ | PHI | corner flow turning angle, deg |
| $2 \theta$ | TH2 | diffuser equivalent cone angle: |
|  |  | $2 \tan ^{-1}\left(\frac{\sqrt{A_{2}}-\sqrt{A_{1}}}{L \sqrt{\pi}}\right), \operatorname{deg}$ |

OF SUBSONIC WIND TUNNEL PERFORMANCE
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SUMMARY

This report brings together and refines the previously scattered and oversimplified techniques for the aerodynamic design and loss prediction of the components of subsonic wind tunnels. General guidelines are given for the design of diffusers, contractions, corners, and the inlets and exits of nonreturn tunnels. A system of equations, reflecting the current technology, has been compiled and assembled into a computer program (a user's manual for this program is included) for determining the total pressure losses. The formulation presented is applicable to compressible flow through most closed- or openthroat, single-, double-, or non-return wind tunnels. A comparison of estimated performance with that actually achieved by several existing facilities produced generally good agreement.

## INTRODUCTION

In the past, most of the work on the design of ducts and wind tunnels and on the determination of their pressure (and power) losses has been either highly specialized, considering only one type of component, or over-simplified, covering several types of components but giving only a superficial idea of what parameters are important. However, for the recent NASA studies directed toward new and modified wind tunnel facilities, it has been necessary to do a careful job of estimating, easily and quickly, the performance of all circuit components. This report brings together, revises, and updates the techniques for the aerodynamic design and performance prediction of subsonic wind tunnels.

The basic procedures and guidelines for the aerodynamic design of critical wind tunnel components, as presented in references 1 through 3, have been revised and updated, as required. The diffuser and contraction design curves developed and suggested herein show the relative design points for several existing facilities. Also provided are recommendations derived from recent NASA studies on end treatments for non-return wind tunnels.

The method of loss analysis presented is a synthesis of theoretical and empirical techniques. Generally, the algorithms used were those substantiated by experimental results. The methods of references 4 through 11 for predicting component losses have been refined and incorporated. The performance
calculations, based on user-selected flow conditions at the test section, assume that the circuit geometry has been predetermined.

The comparison of the actual and predicted performance for several existing wind tunnel facilities shows generally good agreement.

## CAUTIONARY DESIGN GUIDELINES

This report presents the means for rapidly estimating the performance of a wind tunnel circuit after its geometry has been determined. However, an improper design of any of its several components (diffusers, contractions, or corners, for example) could result in performance penalties caused by interaction with the flow in other components; such penalties cannot be predicted. In addition, improper design could cause poor test-section flow quality which would not be indicated by the performance analysis. Therefore, the purposes of this section are to point out critical areas of concern in wind tunnel design and to attempt to establish proper design criteria.

## Diffusers

Diffusers, especially those just downstream of high-speed sections, are very sensitive to design errors which may cause flow separation. The equivalent cone angle and area ratio must be properly selected to avoid steady-state or intermittent separation of the flow from the diffuser walls. (This separation can cause vibration, oscillatory fan loading, oscillations in test section velocity, and higher losses in downstream components.) Generally, proper diffuser design requires that, for a given area ratio, the equivalent cone angle be constrained below a certain value. ("Equivalent" denotes an imaginary conical section with length and with inlet and exit areas identical to the actual section.) This cone angle should probably be held $0.5^{\circ}$ to $1^{\circ}$ lower for diffusers with sharp corners than for those with a rounded cross section.

Since the portion of the wind tunnel between the test section and the fans is usually the higher-loss segment, it is the most critical in affecting circuit performance. Therefore, it was used as a basis for establishing recommended design limits as a guide to diffuser selection. It was assumed that the fans serve to reenergize the boundary layer of downstream sections and that the fans and the upstream and downstream components have no interaction that affects their losses; this may or may not be true (see ref. 12). The overall area ratio and cone angle between the test section and fan contraction were examined for several wind tunnels. This analysis used the centerline lengths of all intervening components, including corners. (The actual effect of corners is unknown: they may alter the onset of separation somewhat.) Figure 1(a) compares curves for the first appreciable stall for flows with thin inlet boundary layers, from references 1 and 2 , with the design points of selected existing wind tunnels. These curves were used to aid in defining the separation trend; good correlation with the symbols is not necessarily expected. Figure $l(a)$ shows that most of these wind tunnels were designed beyond (above) the two-dimensional stall curve but below the conical stall
curve. (Some of these diffusers are far from conical.) The recommended design region, shown in figure $l(\mathrm{~b})$, was positioned with the prior knowledge that the NASA-Ames 7 - by 10 -Foot Wind Tunnel has a partially separated diffuser just downstream of the test section, and that the NASA-Ames 40 - by 80 -Foot Wind Tunnel has some local separation in the corners of the primary diffuser. The upper portion of the design region is recommended for diffusers with rounded corners, and the lower portion for diffusers with sharp corners.

## Contractions

Contracting sections are subject to separation in the same manner as diffusers; however, the penalties are usually much less severe in the contracting sections. Separation of the flow can occur if the contraction is too short for the amount of area reduction. Figure 2(a) presents the general wall shapes suggested in reference 3 and figure 2 (b) shows the design boundary for these shapes in comparison with the designs of several selected wind tunnel facilities.

From this comparison it is evident that, while some facilities were designed more conservatively than others, no design severely exceeds the design boundary. Since none of the facilities considered has shown significant contraction-caused flow problems, the design boundary may be considered empirically reasonable. Further, reference 13 generally tends to support the positioning of the suggested design curve. However, the criteria of reference 13 are more conservative due to consideration of viscous effects which were neglected in the study of reference 3 .

## Corners

The corner losses in a wind tunnel can be large. To minimize them, turning vanes should be used for more efficient turning. Also, as with any other high-loss item corners should, where possible, be located in a large-area section where the flow speed is low. Corner vane losses can be minimized in two additional ways: (1) by selecting an efficient vane cross-sectional shape and adjusting it for proper alignment with the flow, and (2) by choosing the best chord-to-gap spacing.

With reference to item (1), turning vane shapes can vary from bent plates to highly-cambered airfoils. Some sources favor airfoil vanes as being more efficient (ref. 4, p. 63) while others claim that thin vanes can have lower losses (ref. 5, p. 93). But airfoil vanes with blunted leading edges may be more forgiving of misalignment with the flow. The thicker vanes may, therefore, hold some advantage.

When considering item (2), the best chord-to-gap ratio depends on the vane type. For thick vanes, a ratio of about $2.5-$ to-1 is recommended (ref. 4, p. 62) and for thin vanes a ratio of about 4 -to-1 is suggested (ref. 5, p. 92).

Non-return wind tunnels have presented some interesting problems in tunnel design. This type of wind tunnel has the advantages of less structure (and therefore lower construction costs) and of no exhaust-gas-purging or airexchange requirement. Careful design can make the non-return circuit operating power competitive with that of closed-return wind tunnels (the corner losses can be traded for inlet and exit losses). However, an area of concern for the non-return tunnel is its potential sensitivity to external winds which could affect both the required power and the test section flow quality.

A recent series of NASA studies, which dealt with wind s.nsitivity problems, showed that a non-return wind tunnel should have three eatures: (1) a vertical exit system, (2) a horizontal inlet, and (3) an enclused area of protection, with a solid roof, at the inlet. References 14 and 15 detail the development work for the end treatment considered in those studies.

Reference 16 describes an inlet geometry that was developed to reduce the effects of wind. (This reference also presents a set of test-section flowquality requirements by which the characteristics of any inlet treatment may be evaluated.) Although the end treatment designs shown in references 14 through 16 could be revised or refined for additional wind protection, any additional inlet treatment would increase the structural cost and could increase the power requirement.

## PERFORMANCE ESTIMATION

Although the performance analysis presented in this report was systematized and automated for rapid calculation of numerous cases or iterations (by the computer program described in the following sections), the equations presented are equally amenable to manual calculation methods.

## General Approach

The equations were derived in forms that use the most common and convenient defining parameters. The equations are 1 isted and explained below and may be used for component after component, each in turn.

The total pressure losses (proportional to power losses) of each component are calculated and summed to give the total circuit loss and operating power required. The computation technique is applicable to either closed- or non-return circuit types made up of any combination of standard wind tunnel components in any order. The flow conditions in the test section (velocity, and stagnation temperature and pressure) and the external atmospheric pressure are variable as required.

## Problem Restrictions

Three restrictions were found to be necessary in order to allow rapid solution of most cases with a minimum amount of effort. First, the crosssectional geometries were limited to the most common shapes: circular, rectangular, and flat-oval (semi-circular side walls with flat floor and ceiling). Second, air exchangers were omitted from this analysis due to lack of uniformity of configuration and a lack of definition as to the proper method of computing the losses. Finally, the drive system was assumed to be located in one or more parallel, annular ducts.

## $\therefore$ <br> Computation Formulas

The equations used in this performance analysis were synthesized from various sources. Some were used in their original (source) form and others were modified to make them more convenient for use in this analysis. The equations used are presented below.

Flow-state parameters- The basic flow-state parameters were determined from input information about the reference control station and the test section. These parameters were derived from standard relationships for compressible flow.

$$
\begin{array}{cc}
\rho_{T}=\frac{P_{T}}{R T_{T}} & \text { (ref. 17, p. 8) } \\
a_{T}=\sqrt{\gamma R T_{T}} & \text { (ref. 17, p. 51) } \\
\mu_{T}=\frac{\mu_{S T D}\left(\frac{T_{T}}{T_{S T D}}\right)^{0.76}}{\sqrt{1+\left(\frac{\gamma-1}{2} M_{o}^{2}\right)}} & \text { (ref. 18, p. 19) } \\
\rho_{0}=\frac{\rho_{T}}{\left[1+\left(\frac{\gamma-1}{2} M_{o}^{2}\right)\right]^{\frac{l}{\gamma-1}}} & \text { (ref. 18, p. 4) } \\
A_{*}=M_{o} A_{o}\left\{\frac{\gamma+1}{\left.2\left[1+\left(\frac{\gamma-1}{2} M_{o}^{2}\right)\right]\right\}}\right\}^{\frac{\gamma+1}{2(\gamma-1)}} & \text { (ref. 18, p. 4) }
\end{array}
$$

Local conditions- The local flow conditions were determined for each end of each section.

1. Mach number: The local Mach number was found from a Newton's-method solution of the relationship

$$
M^{2}-\left[\frac{\gamma+1}{\gamma-1}\left(\frac{A}{A_{*}} M\right)^{\frac{2(\gamma-1)}{\gamma+1}}\right]+\frac{2}{\gamma-1}=0
$$

(appendix A)
2. Reynolds number: The Reynolds number based on the characteristic length $\ell$, usually the local hydraulic diameter, was determined from

$$
\mathrm{RN}=\frac{\rho_{\mathrm{o}} \mathrm{~V}_{\mathrm{o}} \ell}{\mu_{\mathrm{T}}}\left(\frac{\mathrm{~A}_{\mathrm{O}}}{\mathrm{~A}}\right)\left[1+\left(\frac{\gamma-1}{2} \mathrm{M}^{2}\right)\right]^{0.76}
$$

3. Friction coefficient: A Newton's-method solution was used to determine the friction coefficient for smooth walls from the expression

$$
\left[\log _{10}\left(\lambda \mathrm{RN}^{2}\right)-0.8\right]^{-2}-\lambda=0
$$

Section pressure Zosses- The loss in total pressure caused by each section was calculated in a form non-dimensionalized by local dynamic pressure: $\mathrm{K}=\Delta \mathrm{p}_{\mathrm{T}} / \mathrm{q}$. (In this study the smallest-area end of each section was used as the local reference position.) The individual losses were based on the nature of the section, local flow conditions, and input geometry and parameter information. The most appropriate loss forms for typical wind tunnel sections are catalogued on the following pages. The nonstandard formulas, those which are not directly attributable to the literature, are developed in appendix A. The precise equations, which were developed from various curve-fitting and interpolation techniques based on the plots presented in certain figures, are given in appendix $B$.

1. Constant-area ducts: For closed, constant-area sections the pressure loss due to friction is given by

$$
\mathrm{K}=\frac{\lambda \mathrm{L}}{\mathrm{D}_{\mathrm{h}}}
$$

(ref. 7, p. 53)
2. Open-throat duct: The losses from an open-throat test section may be found from the expression

$$
\begin{equation*}
\mathrm{K}=0.0845 \frac{\mathrm{~L}}{\mathrm{D}_{\mathrm{h}}}-0.0053\left(\frac{\mathrm{~L}}{\mathrm{D}_{\mathrm{h}}}\right)^{2} \tag{ref.7,p.150}
\end{equation*}
$$

3. Contractions: In contracting sections, where the major part of the losses is due to friction, the local loss may be approximated as

$$
K=0.32 \frac{\lambda L}{D_{h}}
$$

(ref. 6, p. 528)
4. Corners with no net area change ("constant area"): A duct can change direction with or without the aid of flow guide vanes. For a constant-area turn employing turning vanes for efficiency, with a "normal" number of vanes (ref. 7, p. 241), and with chord-to-gap ratios between $2-$ to-1 and 4-to-1, the losses resulting from friction and rotation caused by the vanes are

$$
\mathrm{K}=\frac{\mathrm{K}_{\mathrm{TV}}}{3}\left[2+\left(\frac{\log _{10} \mathrm{RN}_{\mathrm{REF}}}{\log _{10} \mathrm{RN}}\right)^{2.58}\right]
$$

The Reynolds number used for the turning vane loss should be based on vane chord. The turning vane loss parameter $\mathrm{K}_{\mathrm{TV}}$ is plotted as a function of turning angle in figure $3(\mathrm{a})$, with the assumption that $\mathrm{K}_{\mathrm{TV}}=0.15$ is a reasonable value for a $90^{\circ}$ corner. Corners without turning vanes are less efficient and the loss function may be approximated by a sixth-order polynomial as shown in figure 3(b):

$$
\begin{aligned}
K= & 4.313761 \times 10^{-5}-6.021515 \times 10^{-4} \phi+1.693778 \times 10^{-4} \phi^{2}-2.755078 \times 10^{-6} \phi^{3} \\
& +2.323170 \times 10^{-7} \phi^{4}-3.775568 \times 10^{-9} \phi^{5}+1.796817 \times 10^{-11} \phi^{6}
\end{aligned}
$$

(appendix B)
This function assumes a loss value of about $K=1.8$ for a $90^{\circ}$ turn. The foregoing losses are those associated with the turning of the flow only. The losses for a corner system (with or without vanes), with the walls of the duct to be considered as well, requires an additional term for the frictional loss of the constant-area duct based on the centerline length.
5. Corners (diffusing): Corners with diffusion may well employ longer vanes in order to improve the efficiency of the diffusion process. For this reason they were treated as vaned diffusers with the addition of the rotational loss term of the turning vane function:

$$
\mathrm{K}=\left\{0.3+\left[0.006\left(2 \theta-21.5^{\circ}\right) \mathrm{u}\left(2 \theta-21.5^{\circ}\right)\right]\right\}\left(\frac{\mathrm{AR}-1}{\mathrm{AR}}\right)^{2}+\frac{2}{3} \mathrm{~K}_{\mathrm{TV}}
$$

(appendix A)
where $u\left(2 \theta-21.5^{\circ}\right)$ is the unit step function and the turning vane loss parameter is defined as for a constant-area corner. This loss function includes the effects of friction.
6. Diffusers: Diffusion produces both expansion and friction losses in the duct given by

$$
K=\left[K_{E X P}+\left(\frac{\lambda}{8 \sin \theta} \frac{A R+1}{A R-1}\right)\right]\left(\frac{A R-1}{A R}\right)^{2}
$$

where the expansion parameter values, $K_{E X P}$, are plotted against equivalent cone angle in figure 4 and the technique used for estimating the $K_{E X P}$ values is described in appendix $A$.
(It should be noted that there are more sophisticated techniques for estimating diffuser performance than the one presented here. However, they require boundary-layer calculations; for example, see reference 19. Experience with both the simple technique described herein and more complex techniques indicates that the two produce comparable results. Generally, little is gained by the significant additional effort required to use the more complex approaches.)
7. Exit: The total pressure loss at the exit of a non-return wind tunnel, or of any expelled flow, is due to the loss of the kinetic energy of the exiting flow. This is given by

$$
K=\frac{2\left\{\left[1+\left(\frac{\gamma-1}{2} M^{2}\right)\right]^{\frac{\gamma}{\gamma-1}}-1\right\}}{\gamma M^{2}}
$$

(appendix A)
8. Fan (power) section: Fan drive sections are commonly made up of contractions, constant-area annular ducts, and diffusers. Analysis should be handled by dividing the fan section into these three component parts.
9. Flow straighteners - honeycomb (thin walls): The loss through thin flow-straightener or honeycomb systems may be expressed as

$$
K=\lambda\left(3+\frac{L}{D_{h}}\right)\left(\frac{A}{A_{\text {FLOW }}}\right)^{2}+\left(\frac{A}{A_{\text {FLOW }}}-1\right)^{2} \quad(\text { ref. 7, p. 478) }
$$

where the hydraulic diameter is that of the honeycomb cell. The friction coefficient is determined from a Reynolds number based on the surface roughness of the honeycomb:
for $R N \leq 275$,

$$
\begin{aligned}
& \lambda=0.375 \mathrm{RN}^{-0.1}\left(\frac{\Delta}{D_{\mathrm{h}}}\right)^{0.4} \\
& \lambda=0.214\left(\frac{\Delta}{\mathrm{D}_{\mathrm{h}}}\right)^{0.4}
\end{aligned}
$$

10. Flow straighteners - airfoil members (thick walls): Flow through adjacent airfoils will first contract and then diffuse. It was assumed that the point of minimum distance between parallel members would be at 30 percent of the straightener length back from the leading edge. The forward 30 percent was treated as a contraction and the aft 70 percent as a vaned diffuser. Thus,

$$
\mathrm{K}=0.096 \frac{\lambda \mathrm{~L}}{\mathrm{D}_{\mathrm{h}}}+\left\{0.3+\left[0.006\left(2 \theta-21.5^{\circ}\right) \mathrm{u}\left(2 \theta-21.5^{\circ}\right)\right]\right\}\left(\frac{\mathrm{AR}-1}{\mathrm{AR}}\right)^{2}
$$

(appendix A)
where the hydraulic diameter is that of each cell of the flow straightener, the friction coefficient is determined from a Reynolds number based on that hydraulic diameter, the area ratio and equivalent cone angle are based on the exit and minimum flow areas, and $u\left(2 \theta-21.5^{\circ}\right)$ is the unit step function.
11. Internal flow obstruction - drag item: The loss due to the drag of internal structure such as struts or models has the form

$$
K=C_{D} \frac{S}{A_{F L O W}} \varepsilon
$$

(appendix A)
12. Perforated plate: Perforated plate with sharp-edged orifices, used as protection screen or as screen around the inlet of a non-return tunnel, produces losses given by

$$
K=\left\{\left[\sqrt{\frac{1}{2}\left(1-\frac{A_{F L O W}}{A}\right)}+\left(1-\frac{A_{F L O W}}{A}\right)\right] \frac{A}{A_{F L O W}}\right\}^{2} \quad(\text { ref. } 7, \text { p. 321) }
$$

13. Mesh screen: The losses produced by a mesh screen may be expressed as

$$
K=K_{R N} K_{\text {MESH }}\left(1-\frac{A_{F L O W}}{A}\right)+\left(\frac{A}{A_{F L O W}}-1\right)^{2} \quad \text { (ref. 7, p. 308) }
$$

where the Reynolds number influence factor, $\mathrm{K}_{\mathrm{RN}}$, is plotted against Reynolds number (based on mesh diameter) in figure 5, and the mesh constant, $\mathrm{K}_{\text {MESH }}$, is 1.3 for average circular metal wire, 1.0 for new metal wire, and 2.1 for silk thread.
14. Sudden expansion: For a sudden expansion with ducting downstream (to allow reattachment of the flow and maximize the pressure recovery) the loss is

$$
\mathrm{K}=\left(\frac{\mathrm{AR}-1}{\mathrm{AR}}\right)^{2}
$$

(ref. 7, p. 128)
15. Vaned diffusers: The pressure loss of a vaned diffuser, one in which splitter vanes are used to improve the performance of a short diffuser by decreasing the effective equivalent cone angle of each chamber, may be determined from

$$
\mathrm{K}=\left\{0.3+\left[0.006\left(2 \theta-21.5^{\circ}\right) \mathrm{u}\left(2 \theta-21.5^{\circ}\right)\right]\right\}\left(\frac{\mathrm{AR}-1}{\mathrm{AR}}\right)^{2}
$$

(appendix A)
where $u\left(2 \theta-21.5^{\circ}\right)$ is the unit step function. (See fig. 6.)
16. Fixed, known loss: For a fixed loss item where the pressure loss value is known, that value may be used directly by definition:

$$
\mathrm{K}=\frac{\Delta \mathrm{p}_{\mathrm{T}}}{\mathrm{q}}
$$

17. Multiple ducts: In a system of multiple ducts, where the local flow passes through two or more separate, identical passages at the same time, the losses have the same value as those for the same type of single duct. Some of the pertinent parameters, such as hydraulic diameter and equivalent cone angle, should be based on the geometry of one of the individual ducts. The loss for the system of ducts may then be determined from the loss for a single duct:

$$
\mathrm{K}=\frac{\Delta \mathrm{p}_{\mathrm{T}_{\text {SYSTEM }}}}{\mathrm{q}}=\frac{\Delta \mathrm{p}_{\mathrm{T}_{\text {DUCT }}}}{\mathrm{q}}
$$

18. Loss value transferred to reference location: Each local loss parameter is calculated basel on local conditions at the smallest-area end of each section and may then be referenced to the test section conditions by the formula

$$
K_{o}=K\left[\frac{A_{O} M}{A M_{O}} \sqrt{\frac{1+\left(\frac{\gamma-1}{2} M_{o}^{2}\right)}{1+\left(\frac{\gamma-1}{2} M^{2}\right)}}\right]
$$

(appendix A)
19. Overa11 and summary performance: The energy ratio of the wind tunnel under consideration is given by

$$
E R=\frac{1}{\sum_{i=1}^{N} K_{o_{i}}}
$$

(ref. 4, p. 69)

The pressure difference across the wind tunnel walls, determining the minimum required structural strength for each section, is given by

$$
\Delta \mathrm{P}_{\mathrm{W}_{\mathbf{i}}}=\mathrm{p}_{\mathrm{T}_{\mathrm{ATM}}}-\left\{\frac{\mathrm{P}_{\mathrm{SC}}-\left(\mathrm{q}_{\mathrm{o}} \sum_{j=1}^{i} \mathrm{~K}_{\mathrm{o}_{j}}\right)}{\left[1+\left(\frac{\gamma-1}{2} \mathrm{M}_{\mathrm{i}}^{2}\right)\right]^{\frac{\gamma}{\gamma-1}}}\right\}
$$

The power required to be input into the flow in order to drive the flow through the wind tunnel at a specified test section speed is expressed as

$$
\mathrm{P}_{\text {INPUT }}=\frac{\left(\sum_{i=1}^{N} \mathrm{~K}_{\mathrm{o}_{i}}\right) \rho_{o}{ }^{2} \mathrm{~A}_{\mathrm{o}} \mathrm{~V}_{\mathrm{o}}{ }^{3}}{2 \rho_{\mathrm{F}}}
$$

The actual drive power required is dependent on the efficiency of the fan/motor system:

$$
\mathrm{P}_{\text {REQUIRED }}=\frac{\mathrm{P}_{\text {INPUT }}}{\eta_{F}}
$$

## COMPUTER PROGRAM DESCRIPTION

The computer program was written in FøRTRAN IV language. It consists of a main program which calls five subroutines and/or six library routines, as required. Two of the subroutines are optional and may be abbreviated and simulated in order to save execution time and/or memory storage space.

## Method of Solution

The general technique used is outlined in the computer program functional flow chart in figure 7. The program was developed in six functional units: a main program and five specialized subroutines. The main program retains general control over the computational flow and calls the subprograms as required.

In the main portion (designated PERFøRM), at first entry into the program, various section-shape geometry relationships and certain semi-empirical diffuser, turning vane, and honeycomb loss functions are defined. The case title card is read and checked for validity by specified code. The tunnel master data control card is then read, checked for validity, and checked for content of pertinent data by the data-checking subroutine. If any errors are found in either of these two preliminary cards, error messages will be printed. Although detected errors will not abort the computer run (unless a card of improper format is encountered where not expected), the case under current consideration will not be computed - only the checking of input errors will then be performed on each section card. Prior to reading the section cards, the units of measure (International System or U.S. Customary System) to be used for the particular case are read. These units of measure are used as the basis for the development of the appropriate flow parameters and test section conditions.

The section cards are read and operated on one at a time. They are checked for validity and input errors by the data-checking subroutine (called DATACK) and the input information, if sufficiently complete, is then used in the computation of the section upstream- and downstream-end geometries. Adjustments to these geometry calculations are made for any multiple-ducted sections. For diffusing sections where the expansion loss parameter was not input by the user, that parameter is generated from predefined functions. Branching of the computational flow then transfers control to the appropriate block of instructions for the remainder of the calculations which are peculiar to the particular section under consideration.

After all section cards have been read in and operated on, each in turn, a case termination card is encountered. The termination card specifies the optional summary operations to be performed. The encounter of this card
signals the end of a case and triggers the final calculations. The codes contained on this card determine the printing of velocity-optimizing and circuit summary information, the plottirg of the summary information, and the return to the beginning for another case.

The data-checking subroutine evaluates the master and section input cards for completeness of data (based on the requirements for the type of section). Then, if any error was detected during computation of a case or if the appropriate termination code was specified by the user, the complete set of input data is tabulated. Messages about errors, omissions, or superfluous information are included.

The subroutine SPEED computes the local Mach number based on local crosssectional area and determines the local flow velocity.

The subroutine FRICTN calculates the local Reynolds number, usually based on the local duct hydraulic diameter, and the local friction coefficient for smooth pipes.

The subroutine ØUTPUT accepts the calculated section parameters along with the section type codes describing the types of information to be output and prints the section information according to the appropriate format.

The subroutine PLøTIT plots the summary information (cummulative total pressure losses and/or wall pressure differential) versus circuit centerline length if requested. The plot is scaled for centimeter or inch plot paper, determined by whether International or U.S. Customary units are used for computation.

The program is terminated after the last operations on a case for which a no-return instruction on the termination card was given by the user.

## Computing Equipment Required

Hardware and machine components- Although this program was written for use on an IBM $360 / 67$ with TSS Monitor, batch mode, an attempt was made to keep it compatible with any system that uses FøRTRAN IV. No magnetic tapes were used. In this version, input is made by cards and the data to be plotted are stored on a disc file for plotting at a later time in an off-line mode. However, it is possible to use a typewriter-type terminal for conversational or real-time computation, typing the data by card-image format, and plotting immediately after the computation has been completed for a case.

The total core required for compilation on the IBM $360 / 67$ was approximately 82000 (decimal) bytes. If necessary this figure can be reduced by eliminating two subroutines, DATACK and PLDTIT. The sizes of the main program and of each subroutine were approximately as follows:

| PERFøRM | 38800 bytes |
| :--- | ---: |
| DATACK | 25700 |
| SPEED | 800 |
| FRICTN | 900 |
| ØUTPUT | 12900 |
| PLDTIT | 2900 |

The program was executed on an IBM 360/67, writing plot data on a disc, logical unit 10. Later the data file was accessed from a 14.8 character-persecond binary-coded-decimal terminal and plotted on a Zeta plotter with $0.005-i n$. step increments. The plot page size was programmed not to exceed 25 by 38 cm (or 10 by 15 in.$)$.

Software- This program was written for use on any computer with sufficient core and with a standard FøRTRAN IV compiler.

The Zeta plotter routines, with minor exceptions, are compatible with the Calcomp routines. The subroutines AXIS, FACT $\emptyset$ R, LINE, PL $\emptyset$ T, SCALE and SYMB $\emptyset \mathrm{L}$ are alike in both Calcomp and Zeta plotting.

CALL AXIS - draws the axis line and annotates the divisions at every two centimeters or each inch (depending on the units of measure specified).

CALL FACTøR - enables the user to produce normal size drawings with plotters which have either 0.01 - or 0.005 -in. increment size. The variable FACT must be set to 1.0 for 0.01 -in. increments and to 2.0 for $0.005-\mathrm{in}$. increment plotters.

CALL LINE - plots centered squares connected by straight 1ines through the coordinate pairs of data values.

CALL PLDT - is used to establish a new point of origin for the pen and paper movements. Before plotting commences, the pen must be positioned where desired along the $X$-axis. The program will position it along the Y-axis. The plot-page size is defined by the values of YLEN and XLEN which are equated to 25 and 38 cm or 10 and $15 \mathrm{in.}$, as required.

CALL PLøTF - is an alternate plotting initialization routine which is available in the Zeta but not Calcomp plot package. It is used in place of PLDTS whenever deferred plotting is desired. The first argument in the call statement indicates the speed of the terminal with which the plotter is interfaced. The second argument is the logical-device number of the plot file.

CALL SCALE - examines the data and determines the proper scaling for the given dimensions of the plotter paper, 25 by 38 cm or 10 by 15 in .

CALL SMøDE - is available only in the Zeta plot package. It permits the user to choose from extensive capabilities which affect several of the plotter routines. In this program the options have been set equal to the usage found in the Calcomp routines, and therefore, if Zeta plotter routines are not available, the call to SMøDE should be eliminated.

CALL SYMBøL - prints the input case title at the top left of each plot page as it appears in columns 2 through 80 of the title card. For reference purposes, it also draws a small plus sign at the origin of the plot.

The library routines used are standard F $\emptyset$ RTRAN routines:
ABS - Absolute value
ALØG10 - Common logarithm, base ten
ATAN - Arctangent (result in radians)
EXP - Exponential of the natural number $e$
IFIX - Convert from real number to integer
SIN - Trigonometric sine (argument in radians)
SQRT - Square root

## Programming Techniques

It was intended that this program be usable on as many different computer systems as possible. Therefore, in order to make them applicable to some machines, certain statements were forced into particular forms which would be less efficient on other systems (e.g., Hollerith instead of literals in format statements).
$C \emptyset M M \emptyset$ and $D A T A$ statements were used as much as possible to simplify the definition of parameter values. In the main program, arithmetic statement functions were used for three purposes: (1) for the definition of section hydraulic diameter, area, and equivalent cone angle geometry functions; (2) for the conversion function from local to reference-section pressure losses; and (3) for the definition of the least-squares-polynomial-curve-fit functions. The last group of functions includes: (1) the corner turning-loss parameters as functions of turning angle (see fig. 3); (2) the diffuser expansion loss parameters for the different cross-section shapes as functions of equivalent cone angle (see fig. 4); and (3) the mesh screen Reynolds number sensitivity factor as a function of mesh-diameter-based Reynolds number (see fig. 5).

Certain functions not easily solvable in closed form were solved iteratively (some by Newton's method) to 0.01 percent accuracy. These functions include test section Mach number, local section Mach number, and local section friction coefficient.

Numeric codes were used for specifying such things as section type, section end-shape types, and system of computational units; for decisions on requirements for inputs to each section type; and for case-termination procedures and outputs desired. The various important input codes are listed in tables 1 and 4. All sections of the multiple-ducted type were assigned high code numbers for simplicity in selecting them for special handling. The
various section types were grouped in code decades for reasonable association of section code and component function. Where possible, the second digit of the code (if that second digit is not zero) reflects the basic characteristic of the section: constant area, contracting, or diffusing.

The information input fields on the master data and section cards were arranged in three basic groups: (1) qualitative information (type and shape); (2) quantitative geometry information (number of ducts, cross-sectional end dimensions, and length); and (3) loss-related parameters. The case termination card employs the same format as the section cards so that it may be encountered at random intervals without causing a program crash. For the tabulation of the input data (for error-location and record-keeping purposes), objecttime formatting was used to compile the combination input and annotation data set for a convenient output.

Much of the output of the program was set up on a demand (i.e., optional) basis. A section-by-section performance analysis is automatically provided. A brief summary of the variation of selected parameters through the circuit, and plots of those parameters, may be selected if desired. An annotated listing of the inputs may be requested or, if errors are detected, the listing is internally forced in order to provide a simple means of error-detection and correction and/or simplified record-keeping of case data.

## Source Code

A source code listing of the performance estimation program is provided in appendix $C$ along with the associated notation definitions. The source code includes the use of comment cards throughout the program for identification of the operations carried out by each set of instructions.

## Operating Instructions

The basic source program deck arrangement is shown in figure 8.
Input- Sample coding forms for the four types of input cards required are presented in figure 9. The special symbols required in the first columns of the title and master data cards are included.

1. Title card: For the title labeling card, with the exception of the first column which must contain an asterisk (*), the entire card may be used as desired. This title was programmed to appear at the top of each page of the case to which the title refers, including the plots. Only one title card per case may be used.
2. Master card: The tunnel master control data card provides sufficient information for defining conditions in the test section (which is the reference section for all calculations) and conditions of the surrounding external atmosphere. Table 2 details the inputs included on the master data card. The first column must contain a minus sign in order to identify the card as a valid master card. The remainder of the inputs should be positive, with
columns 2 through 6 containing five fields of integers only (no decimal points). Columns 7 through 10 were not used on this card and should be left blank. Columns 11 through 50 should contain floating-point numbers. These columns were divided into eight parameters of five columns each, including decimal point.
3. Section cards: The individual section information cards were based on the same format as the master card, except that the section cards require no special identifying code. Table 3 details the inputs contained on the section cards. The first six columns, containing four data fields, require integer inputs. The remaining 74 columns were divided into two real number fields of two columns each (with the assumed default decimal points to the right of the second columns), and 14 real number fields of five columns each (with the assumed default decimal point between the third and fourth columns of each field).

Although the input parameter requirements vary from section to section, certain requirements are basic to all sections. These items include: (1) the section type code, (2) the section end shape codes, and (3) the section dimensions (end height(s) and width(s) and/or diameter(s) and usually length). A detailed list of the additional, specialized requirements for each section is presented in table 4.

Although not mandatory in order to obtain a correct total power estimation, it is advisable to input the section data cards in the actual section order so that the summary calculations and plots have relevance to the actual circuit.
4. Termination card: The case termination card, which signals the end of the section inputs for a particular case, is identified by the constraint of blanks in card columns 3 and 4. The numbers contained on this card are used strictly as task codes; table 5 shows the details of these codes. In the event of a request for plotted information, the code determines the type(s) of information to be presented. For all other tasks the codes dictate a simple yes/no decision.

As many cases as desired may be input in a single job submission. The same system of units need not be used in all cases. Any parameters may be changed as desired from case-to-case since there are no forced carry-overs (except the specific heat ratio, $\gamma$, which is fixed at the time of program compilation).

Output- Based on the foregoing input information the results may be calculated and tabulated in five different types of information groups.

1. Section performance analysis: The section performance tabulation fully describes the performance-related parameters of the wind tunnel circuit. Atmospheric and test section flow reference conditions are stated at the top of the first page. The various parameters are tabulated for each section in the order of computation with the upstream end information on the first and the downstream information on the second of the two lines for each section. The section sequence number and type (a translation of the code) and the end
shapes are given first. The geometry and local velocity information are presented next, followed by the section length and calculated total pressure loss values.
2. Overall performance: If no input errors are encountered during the analysis of a case, overall performance values are presented at the end of the section performance tabulation. This includes the total circuit length, the total pressure losses and energy ratio for the circuit, and the total operating power required.
3. Summary characteristics: If requested on the termination card and barring any errors, a summary of the circuit characteristics is tabulated on a separate page. This tabulation includes section sequence numbers, Mach numbers, cumulative pressure losses, and local wall pressures, all as functions of distance through the circuit.
4. Plots: Under proper condition codes, the cumulative pressure losses and/or the wall pressure differentials will be plotted as functions of distance through the circuit (centerline length). The straight lines that appear on the plots connecting the points are for reference only and do not represent the actual distribution in a component.
5. Input data tabulation: Finally, if an input error was encountered during the analysis of the circuit, or if such information was requested by the user, the input cards are tabulated with annotations regarding missing or superfluous inputs. A careful look at this section should allow the user to discover why a given set of input data did not produce the expected type of results.

A11 of the foregoing types of output are shown for the test case.
Computer system restrictions- Certain restrictions and/or assumptions had to be imposed on the computer system and its methods and abilities in order to perform the performance analysis within reasonable time, effort, and money constraints.

1. Hardware: This analysis was programmed for a moderate-sized system with common components. No special hardware is required with the exception of a plotter if the plotting option is used. The output printing device is assumed to have available a minimum capacity of 120 characters per line, but the number of lines per page may be set by means of the LINEMX parameter in the main program. (Barring any special requirements, 45 lines for an 8.5-in. page or 60 lines for an $11-i n$. page are recommended.)
2. Software: Certain software restrictions were imposed simply as a starting point to the problem solution. The input card formats were fixed as shown in figure 9. The specific heat ratio ( $\gamma$ ) and the number of lines per output page were fixed for each compilation of the source deck, although changes can be made by altering the values of $G$ and/or LINEMX, respectively, near the beginning of the main program.

For reasons of possible memory limitations on smaller systems, the number of wind tunnel components in each circuit case was limited to 30 sections. This limit may be changed by assigning a new value to LMTSEC in the main program and by re-dimensioning the following variables as denoted by "XX": in the main program (PERFØRM), DELP(XX+2), SEKO(XX), SEL(XX), SMACH(XX), SSUMEL (XX+2) and SSUMKO (XX+2) ; in the data-checking subroutine (DATACK), ENDATA(XX,20), NCHECK (XX, 20) and NDATA(XX,4); and in the plotting subroutine (PLØTIT), DELP(XX+2), SSUMEL(XX+2) and SSUMKO (XX+2). If memory limitations are a severe problem and/or if computer-controlled plotting facilities are not available to the user, the data-checking and/or plotting subroutines may be "removed" by inserting dummy, one-card subroutines with the same arguments which would have no effect on the calculations. This would decrease the utility and power of the program, but would retain the basic performance estimation capabilities without crippling them altogether.

The plotting routines were written according to the requirements for a plotter with $0.005-i n$. increments.

Optional inputs- Certain of the parameter inputs are designated as optional and have built-in assumed default values in the event that the user knows no better values than the ones provided in the sources referenced herein. These optional parameters are shown in tables 2 through 4.

On the master card (see table 2), the units of measure should be specified and an error message will be given if they are specified erroneously (other than as type 1 or 2). However, the units code will default to 1 (the International System) and case execution will continue. The test section and atmospheric total pressures will default to one atmosphere if not specified.

On the section cards (see table 3), the number of items in the duct will default to unity if not specified. The expansion loss parameter for diffusers defaults to a value based on figure 4. (It is computed by determining the shape of each end, the extent to which the diffuser is two-dimensional in nature (i.e., changing cross-sectional size in height or width only), and the equivalent cone angle, and then interpolating between the curves of figure 4. See appendix A.) The mesh screen loss constant defaults to 1.3 , the value for an average-condition metal mesh screen (ref. 6, p. 527), and the reference Reynolds number for turning vanes defaults to 0.5 million (ref. 6, p. 527). The surface roughness for honeycombs defaults to the appropriate equivalent of 0.00001 m , the value for new, commercially smooth, non-steel pipe (ref. 7, p. 62). The factor for the additional influence of a blockage on downstream sections ( $\Delta \varepsilon$ ) defaults to zero.

Diagnostic messages- There are a limited number of error diagnostic messages which were built in to handle many, but not all, of the potential user errors. The causes and appropriate corrections of these errors should be evident in each message.

1. Title card: If a card is in the position of a title card and does not begin with an asterisk as required, the following message will appear:

TITLE ('...(invalid title)...') IS INCORRECT OR IMPROPER AS IT EXISTS. THE FIRST CARD COLUMN MUST CONTAIN AN ASTERISK (*) TO BE IDENTIFIED AS A VALID TITLE CARD.
2. Master card: An invalid master card is denoted by:

MASTER CONTROL DATA ('...(card image)...') IS INCORRECT OR IMPROPER AS IT EXISTS. THE FIRST TWO CARD COLUMNS MUST CONTAIN A NEGATIVE NUMBER (-1 TO -9) TO BE IDENTIFIED AS A VALID MASTER CARD. THIS CASE WILL BE SKIPPED.

A general omission from the master card of required information produces:
CRITICAL OMISSION(S) IN TUNNEL MASTER CONTROL DATA PREVENT EXECUTION OF THIS CASE. ANY SUCCEEDING CASES WILL NOT BE AFFECTED.

Two master cards, back-to-back, for a given case are identified by:
MORE THAN ONE MASTER CONTROL CARD EXISTS FOR THIS CASE OR INPUT CARDS ARE OUT OF ORDER. CHECK DECK SET-UP. THE LAST MASTER CARD ENCOUNTERED WILL BE ASSUMED AS THE CORRECT MASTER CARD FOR THE SECTION CARDS WHICH FOLLOW.

Encountering a master card where not expected (generally indicating missing case termination and title cards) causes this message:

MASTER CONTROL CARD HAS BEEN ENCOUNTERED BEFORE CASE TERMINATION AND TITLE CARDS. CHECK DECK SET-UP. ERROR-MESSAGE TITLE WILL BE GENERATED AND SUMMARY OUTPUT, NO-PLOT, INPUT DATA TABULATION, AND NEXT-CASE RETURN TERMINATION PARAMETERS WILL BE ASSUMED.

If an invalid test section upstream end shape geometry is specified, one which the program cannot handle, an error results:
**ERROR -- INVALID TEST SECTION UPSTREAM END SHAPE CODE WAS SPECIFIED AS (code used) (SHOULD BE 1, 2 OR 3). THIS CASE CANNOT BE EXECUTED.

If an invalid units code is specified the message is:
THE UNITS OF MEASURE CODE IS IMPROPERLY SPECIFIED AS (code used), (SHOULD BE 1 OR 2). CHECK MASTER CARD (COLUMN 4). SEE THE DATA TABULATION AT THE END OF THIS CASE. THE INTERNATIONAL SYSTEM OF UNITS WILL BE ASSUMED FOR THIS CASE.

If the termination code requests power-matching but the input power value is such that the calculation would be meaningless, a diagnostic of the following form is printed:
**ALTHOUGH VELOCITY-OPTIMIZING WAS REQUESTED BY TERMINATION CODE, THE INPUT POWER VALUE IS ILLEGAL (LESS THAN OR EQUAL TO ZERO). THEREFORE, NO VELOCITY-OPTIMIZING IS POSSIBLE. RECHECK INPUT VALUE ON MASTER DATA CARD.
3. Section card: A general omission of required data from a section card will cause this message:

```
**ERROR -- CRITICAL OMISSION(S) IN SECTION INPUT DATA. SEE DATA
TABULATION AT END OF OUTPUT FOR THIS CASE.**
```

If an invalid section shape code is specified it is not possible for the program to properly compute section end geometries; as a result an error occurs:

```
**ERROR -- INVALID SECTION SHAPE CODE WAS SPECIFIED AS (input code)
(SHOULD BE 1, 2 OR 3). THIS SECTION WILL BE SKIPPED.
```

An error which arises during computation and causes a non-positive total pressure loss for a given section prevents completion of the case analysis and gives rise to an error message:
**ERROR -- SOME INCORRECT COMBINATION OF INPUTS OR UNANTICIPATED SITUATION HAS CAUSED AN INVALID (NON-POSITIVE) TOTAL LOSS LEVEL. RECHECK SECTION (section number) INPUT DATA.

If the maximum allowable number of circuit sections written into the program is exceeded by placing too many section cards together in one case, or without termination, title, and master cards between cases, this diagnostic will appear:

MAXIMUM LIMIT ON THE NUMBER OF SECTIONS (... (maximum allowable number of section)...) HAS BEEN REACHED. EITHER A CASE TERMINATION CARD HAS been omitted (along with title and master cards to begin a new case) OR THIS CASE IS TOO LONG FOR THE PROGRAMMED ALLOWABLE NUMBER OF SECTIONS. THE CASE HAS BEEN TERMINATED AT THIS POINT.

In this instance, the inputs from the group of sections for which the limit was exceeded will be tabulated and the remaining section inputs will be evaluated and tabulated. If the user fails to cause the test section blockage amounts specified on the master control card to coincide with that of the test section card, erroneous analysis may result since inconsistent flow areas would be calculated. The section card value will be used (since the discrepancy may be desired) and this notice is given:
**NOTE -- TEST SECTION BLOCKAGE FROM SECTION CARD INPUT (... (section input value)... PERCENT) DOES NOT EQUAL THAT OF THE MASTER CARD INPUT (...(master input value)... PERCENT). CHECK DATA DECK. SECTION CARD VAlUE WILL BE ASSUMED AS CORRECT AND EXECUTION WILL CONTINUE.

An invalid section type code will cause a section to be skipped and a message to be printed:
*ERROR -- INPUT SECTION TYPE CODE (CARD COLUMNS 3 AND 4) CALLS INVALID SECTION TYPE. DATA CARD IGNORED.**

Any input errors were deemed justifiable cause for judgment as an incomplete case. As a result, reliable overall and summary information cannot be calculated. To assist the user in locating the error(s), the input values will be forced to be tabulated and the following explanation appears:

***DUE TO ERROR(S) IN INPUT CARD(S), VALID SUMMARY INFORMATION IS NOT AVAILABLE. REFER TO THE TABULATION OF INPUT DATA ON THE FOLLOWING PAGES. CORRECT THE ERROR(S) AND RESUBMIT THIS CASE. SUBSEQUENT CASES WILL NOT BE AFFECTED.

4. Possible errors lacking diagnostics: Certain potential problem areas remain unprotected by diagnostic and error-recovery systems.

No special provision was made for two test sections in the same circuit case. As long as the blockage values for both test sections match the one from the master card, no message will be printed. In any event, the execution will not be terminated. The test section shapes and dimensions from the master card are not checked against those of the test section card. Although a mismatch of these values could cause a mass-flow error, including and enforcing such a check could inhibit any meaningful tandem-test section cases. These problems could be avoided, however, by naming only one working section as a test section and referring to the other by general type.

Also, there was no provision for checking the specified tunnel type against the types of sections actually used (e.g., checking a non-return, or open-test-section tunnel for exit or open-throat test section input cards). This check is not critical and was left to the user.

One error-check was not included due to the program complications which would have resulted. If a case termination card is omitted at the end of a case and a computer-system control card or a title card is encountered, the error will be disastrous due to mismatched format types. Execution and calculations will be immediately aborted by the computer.

Test Case
The NASA-Ames Research Center 40- by 80 -Foot Wind Tunnel was used as an example of a typical wind tunnel. This tunnel, illustrated in figure 10, is of the single-return, closed-test-section, continuous-running type. It has a flat-oval test section 12.2 m ( 40 ft ) high by 24.4 m ( 80 ft ) wide and is powered by six $12.2-\mathrm{m}$ ( 40 ft ) diameter, six-bladed fans. It has an eight-toone overall contraction ratio and uses multiple-circular-arc type turning vanes in each of the four $90^{\circ}$ corners.

A complete list of the test case inputs and computed information outputs are presented in figure 11. The machine computing time for this test case (without plots) was about 7 sec on an IBM 360/67.

Although this test case was not an exhaustive exercise of all possible tunnel components, it does include most of the basic section types: diffusing test section, single-duct contraction and diffuser, constant-area single duct,
constant-area corner with turning vanes, and multiple-duct fan sections (contraction, constant-area annulus, and diffuser). Examples of other types of components are shown in the sample cases which follow.

## DISCUSSION AND APPLICATIONS

Wind tunnel energy ratio, required power, and operating velocity are interdependent. The energy ratio is affected by velocity through the effect of velocity on the Reynolds number. The required drive power, influenced directly by operating velocity and inversely by energy ratio, is also controlled by the fan system efficiency which is often only an estimated quantity. Any estimate of operating velocity for a given power level is, then, dependent on the basic efficiency of the circuit (energy ratio) and drive system efficiency, assuming the best power estimate available to be that delivered to the fans. This interdependency means that an error in the prediction of energy ratio (and/or in the estimation of fan efficiency) will cause corresponding errors in power and velocity estimates.

These errors resulting from an erroneous prediction of the circuit energy ratio can be found from the relationship governing required power, test section velocity, and energy ratio, assuming given motor electrical and fan efficiencies. For a fixed test section velocity,

$$
\frac{\Delta \mathrm{P}_{\text {REQUIRED }}}{\mathrm{P}_{\text {REQUIRED }}}=\frac{1}{1-\frac{\Delta E R}{\mathrm{ER}}}-1
$$

and, for constant power, an error in energy ratio yields the performance penalty

$$
\frac{\Delta \mathrm{V}_{\mathrm{o}}}{\mathrm{~V}_{\mathrm{O}}}=1-\left(1-\frac{\Delta \mathrm{ER}}{\mathrm{ER}}\right)^{1 / 3}
$$

The expected true power and velocity levels can thus be obtained from the performance estimate:

$$
P_{\text {REQUIRED }}=\left(\frac{1}{1-\frac{\Delta E R}{E R}}\right) P_{\text {REQUIRED }}^{\text {Estimate }} \text { }
$$

for a given set of test section conditions, and from

$$
\mathrm{V}_{\mathrm{O}} \approx\left(1-\frac{\Delta E R}{E R}\right)^{1 / 3} \mathrm{~V}_{\text {Ostimate }}
$$

for a given level of required power.

This adjustment of the estimated performance values is pointless for a known, existing wind tunnel, but necessary for new, or proposed facilities. Before the adjustment can be made the probable error in the energy ratio estimate must be determined. It is desirable, therefore, to consider several existing facilities of different circuit types in order to gain a degree of confidence in the performance estimation routine.

## Results

The input parameters and output performance values for the several sample cases, other than the test case shown in figure 11, are compiled in appendix D. The estimated energy ratios for the seven sample wind tunnels are presented in table 6. The corresponding sketches for all these sample tunnel circuits are shown in figures 10 and 12.

The actual energy ratios for the first three wind tunnels presented in table 6 were estimated from the best available information on fan and electrical efficiencies from known input power levels. The actual performance of the other four facilities was taken from measured data.

The test case and first sample case was the circuit of the NASA-Ames Research Center 40- by 80-Foot Wind Tunnel as described previously in the test case discussion. The predicted energy ratio for this rather conventional tunnel was only 1 percent higher than the actual value when new.

The performance of the NASA-Ames 7- by 10-Foot Tunnel was predicted at a slightly optimistic level. However, this tunnel is one with several known problems which complicate the prediction process. With the air exchanger operating, the primary diffuser is known to have some local flow separation, having been designed at a $6^{\circ}$ equivalent cone angle, an angle too great for its cross-sectional shape and length (see fig. 1). Also, the drive fan is stalled from the centerbody out to about 45 percent of the fan radius, causing some back-flow along the nacelle centerbody. (The impact of the stall on the fan efficiency has only been estimated; it was assumed that the fan efficiency would suffer by about an additional 10 percent.) In spite of these things, the predicted energy ratio was only about 3 percent too high relative to the original value of approximately 7.85 , both values taken in the zero-airexchange configuration. This agreement may indicate that much of the abovementioned off-design performance is triggered by the air exchanger operation and is not as significant with the air exchanger closed. Although insufficient data are available to resolve this question, the fact remains that the prediction accuracy, for the stated conditions, was good.

The Lockheed-Georgia Low-Speed Wind Tunnel employs a tandem test section design. For this analysis, the larger, V/STOL test section was used as the only reference station. Because of the two area restrictions to cross sections smaller than that of the reference area (those of the smaller test section and of the fan), the tunnel efficiency would be expected to be low. (This in no way reflects on the tunnel's usefulness as a research tool or on its design or capabilities. The "low efficiency" value results only from the point of reference used in the calculations.) In other than these features
the facility is basically of conventional design. The computerized performance prediction was in error by less than 2 percent from the true value of about 1.10 .

The Indian Institute of Science 14- by 9-Foot Tunnel at Bangalore stands out among non-return wind tunnels as a facility with an unusually high energy ratio. Although the determination of circuit dimensions for the program input was somewhat hampered by the limitations of small drawings, the estimated energy ratio was within 1 percent of the true facility value of 6.85 . It is interesting to note that the fan performance data of reference 23 would indicate a fan design efficiency of about 69 percent. The power requirement calculations based on energy ratio and test section maximum velocity, however, show that the fan efficiency must be higher than was expected; in fact, greater than 90 percent.

The Hawker Siddeley 15-Foot V/STOL Tunnel at Hatfield, England was constructed under economy constraints and is a compact, cost-effective facility. The estimated performance was about 1.6 energy ratios higher (i.e., more optimistic) than the actual value of 2.38 . This is an error of about 67 percent. The primary performance difference was probably caused by the fan system. The losses of the ducting in this area are difficult to predict because the area changes are not gradual and are even difficult to define.

The University of Washington 8- by 12-Foot Double-Return Tunnel has a surprisingly high measured energy ratio of 8.3. This would indicate a very carefully designed circuit powered by carefully designed fans. The performance estimate produced by the computer program is lower than the actual energy ratio by about 13 percent, showing that the achieved performance level is higher than would normally be expected.

The NASA-Langley Research Center 30 - by 60 -Foot Open-Throat Tunnel is unusual in configuration, having a double-return system with the twin fans located less than two fan-diameters downstream of the test section. The location of the data point for this tunnel on the diffuser design curves of figure 1 would not indicate that any diffuser-related problems should be expected forward of the fans. The diffuser between the fans and the first corner, however, does have a rather large equivalent cone angle (more than $8^{\circ}$ ). If the fans cause or contribute to diffuser flow problems (see the Cautionary Design Guidelines for Diffusers) and if those problems lead to corner flow inefficiency in a region critical to overall performance, then the circuit energy ratio may be well below the normal estimated level. Although it is not clear whether this is the case in the NASA-Langley 30 - by 60 -Foot Tunnel, the performance estimate was about 27 percent higher than the actual value of about 3.71.

## Evaluation

To summarize what may be learned from the sample cases:

1. The Ames 40 - by 80 -Foot and 7 - by 10 -Foot Tunnels and the LockheedGeorgia Low-Speed Tunnel, although at opposite ends of the energy ratio
spectrum, are all basically standard, single return, closed-test-section facilities; the computer program estimates of actual performance were good.
2. The Indian Institute of Science Bangalore tunnel, being of the nonreturn variety, is a different and less common type of facility; the computer program closely estimated its actual performance.
3. The University of Washington double-return tunnel is a third major circuit type; the program produced a reasonably accurate prediction of its performance.
4. The Hawker Siddeley V/STOL and Langley 30 - by 60 -Foot Tunnels are examples of facilities which may have flow problems due to too-rapid area changes and, as a result, lower than optimum performance levels for their respective circuit types. For these tunnels, because of their flow quality and not because of their circuit types, the program provided a poor estimate of actual performance.

Based on these results one thing is immediately clear: the performance of a wind tunnel of conventional, conservative design can be evaluated accurately. On the other hand, the performance of a tunnel whose design generates or contributes to flow problems (separation or grossly non-uniform) will be overestimated by the loss equations and computer program.

Flow peculiarities and off-optimum designs, even though seemingly only slight, can cause operational performance to fall significantly below the predicted levels. Such problems can be expensive whether considered in terms of modifying the facility or in such terms as reduced testing capability and increased power costs. Judicious, iterative use of the estimation techniques presented in this report, simplified by computerized automation, can lead to the improvement of an existing facility through guidance of design changes or to the optimization of a proposed new wind tunnel design.

Ames Research Center<br>National Aeronautics and Space Administration Moffett Field, California 94035, January 8, 1976

## APPENDIX A

NON-STANDARD FUNCTIONAL FORMS

Due to the nature of this analysis, certain of the local flow-state, section loss, and summary parameter formulas were used in a form more convenient than that usually found in the literature. The relationships which were altered or derived are outlined on the following pages.

## Local Flow-State Parameters

The calculation of several local parameters was based on the local Mach number, determined from the relationship between the local area and the area for choked flow:

$$
\frac{A_{ \pm}}{A^{\prime}}=\left(\frac{\gamma+1}{2}\right)^{\frac{\gamma+1}{2(\gamma-1)}} M\left[1+\left(\frac{\gamma-1}{2} M^{2}\right)\right]^{-\frac{\gamma+1}{2(\gamma-1)}}
$$

(ref. 18, p. 6)

Solving for the area for choked flow, knowing the test section area and Mach number,

$$
A_{*}=M_{o} A_{0}\left\{\frac{\gamma+1}{2\left[1+\left(\frac{\gamma-1}{2} M_{o}^{2}\right)\right]}\right\}^{\frac{\gamma+1}{2(\gamma-1)}}
$$

Mach number- Another form of the same area relationship,

$$
\begin{equation*}
\left(\frac{A}{A_{*}}\right)^{2}=\frac{1}{M^{2}}\left\{\frac{2}{\gamma+1}\left[1+\left(\frac{\gamma-1}{2} M^{2}\right)\right]\right\}^{\frac{\gamma+1}{\gamma-1}} \tag{ref.17,p.126}
\end{equation*}
$$

can be rewritten to produce a polynomial equation in Mach number which may be solved by Newton's method if the areas are known:

$$
\begin{aligned}
& {\left[\left(\frac{A}{A_{*}}\right)^{2} M^{2}\right]^{\frac{\gamma-1}{\gamma+1}}=\frac{2}{\gamma+1}\left[1+\left(\frac{\gamma-1}{2} M^{2}\right)\right]} \\
& =\frac{2}{\gamma+1}+\left(\frac{\gamma-1}{\gamma+1} M^{2}\right) \\
& M^{2}-\left[\frac{\gamma+1}{\gamma-1}\left(\frac{A}{A_{*}} M\right)^{\frac{2(\gamma-1)}{\gamma+1}}\right]+\frac{2}{\gamma-1}=0
\end{aligned}
$$

Reynolds number- The local Reynolds number was calculated based on other, known, local conditions and from basic principles:

$$
\begin{align*}
& \mathrm{RN}=\frac{\rho \mathrm{V} \ell}{\mu} \\
& \rho \mathrm{VA}=\rho_{\mathrm{o}} \mathrm{~A}_{\mathrm{o}} \mathrm{~V}_{\mathrm{o}} \text { (conservation of mass) } \\
& \mu=\mu_{\mathrm{T}}\left(\frac{T}{T_{T}}\right)^{0.76} \text { (ref. 18, p. 19) }  \tag{ref.18,p.19}\\
& \frac{T}{T_{T}}=\left[1+\left(\frac{\gamma-1}{2} M^{2}\right)\right]^{-1} \text { (ref. 18, p. 4) } \\
& \mathrm{RN}=\frac{\rho_{0} V_{0} \ell}{\mu_{T}} \frac{A_{0}}{\mathrm{~A}}\left[1+\left(\frac{\gamma-1}{2} M^{2}\right)\right]^{0.76}
\end{align*}
$$

Friction coefficient- The Reynolds number-friction coefficient function used was

$$
\begin{equation*}
\frac{1}{\sqrt{\lambda}}=2 \log _{10}(\operatorname{RN} \sqrt{\lambda})-0.8 \tag{ref.6,p.70}
\end{equation*}
$$

A Newton's method solution was performed on a rewritten form of the equation:

$$
\left[\log _{10}\left(\lambda R N^{2}\right)-0.8\right]^{-2}-\lambda=0
$$

## Section Pressure Losses

The losses for some types of sections were derived in forms not found in the literature. For others, a curve-fit of data points or a simplification of analysis was performed.

Corners (constant area)- The frictional and rotational losses through turning vanes are additive: $K=K_{\text {FRICTION }}+\mathrm{K}_{\text {ROTATION }}$. Also,

$$
K_{\text {FRICTION }}=\frac{1}{3} \mathrm{~K}_{\mathrm{TV}} \text { and } \mathrm{K}_{\text {ROTATION }}=\frac{2}{3} \mathrm{~K}_{\mathrm{TV}} \quad \text { (ref. 6, p. 527) }
$$

Assuming that the frictional loss value has a form similar to that for a flat plate, then at $90^{\circ}$ :

$$
\begin{equation*}
\mathrm{K}_{\text {FRICTION }}=\frac{1}{3} \mathrm{~K}_{\mathrm{TV}}^{90} 10=\frac{0.455 \mathrm{~B}}{\left(\log _{10} \mathrm{RN}\right)^{2.58}} \tag{ref.6,p.527}
\end{equation*}
$$

Thus, the constant $B$ is dependent on the turning vane loss constant and the reference Reynolds number at which that constant was determined:

$$
B=\frac{\frac{1}{3} K_{T V_{90}}\left(\log _{10} \mathrm{RN}_{\mathrm{REF}}\right)^{2.58}}{0.455}
$$

Therefore,

$$
\mathrm{K}_{\text {FRICTION }}=\frac{1}{3} \mathrm{~K}_{\mathrm{TV}}^{90} \text { }\left(\frac{\log _{10} \mathrm{RN}_{\mathrm{REF}}}{\log _{10} \mathrm{RN}}\right)^{2.58}
$$

Since the rotational term is assumed independent of Reynolds number, $K_{\text {ROTATION }}=(2 / 3) \mathrm{K}_{\mathrm{TV}}^{90^{\circ}}$. The additional complication of loss parameter variation with turning angle is presented in figure 3 for a loss parameter at $90^{\circ}$ equal to 0.15 . It was assumed that the relationship between the actual and reference loss constants is 1inear:

$$
\mathrm{K}_{\mathrm{TV}}=\mathrm{K}_{\mathrm{TV}}^{90} \text { }\left[\frac{\mathrm{f}(\phi)}{0.15}\right]
$$

where $f(\phi)$ is the functional relationship plotted in figure 3. The complete turning vane loss function then becomes

$$
\mathrm{K}=\mathrm{K}_{\mathrm{TV}}\left\{\frac{2}{3}+\left[\frac{1}{3}\left(\frac{\log _{10} \mathrm{RN}_{\mathrm{REF}}}{\log _{10} \mathrm{RN}}\right)^{2.58}\right]\right\}
$$

Corners (diffusing)- Diffusing corners were treated as vaned diffusers with the addition of rotational losses dependent on the turning angle. The expansion and frictional losses used were those for a vaned diffuser:

$$
\mathrm{K}_{\text {VANED DIFFUSER }}=\left\{0.3+\left[0.006\left(2 \theta-21.5^{\circ}\right) \mathrm{u}\left(2 \theta-21.5^{\circ}\right)\right]\right\}\left(\frac{\mathrm{AR}-1}{\mathrm{AR}}\right)^{2}
$$

The rotational loss is as for a constant-area corner where

$$
\mathrm{K}_{\text {ROTATION }}=\frac{2}{3} \mathrm{~K}_{\mathrm{TV}}=\frac{2}{3} \mathrm{~K}_{\mathrm{TV}}^{90} 1\left[\frac{\mathrm{f}(\phi)}{0.15}\right]
$$

The diffusing corner loss function is then

$$
\mathrm{K}=\left\{0.3+\left[0.006\left(2 \theta-21.5^{\circ}\right) \mathrm{u}\left(2 \theta-21.5^{\circ}\right)\right]\right\}\left(\frac{\mathrm{AR}-1}{\mathrm{AR}}\right)^{2}+\frac{2}{3} \mathrm{~K}_{\mathrm{TV}}
$$

where $u\left(2 \theta-21.5^{\circ}\right)$ is the unit step function.

Diffusers- The diffuser losses are due to both friction and expansion. The friction term may be derived theoretically from

$$
\mathrm{K}_{\text {FRICTION }}=\left.\frac{\Delta \mathrm{p}_{\mathrm{T}}}{\mathrm{q}}\right|_{\text {FRICTION }}=\int_{0}^{\mathrm{s}_{2}} \frac{\lambda \rho \mathrm{~V}^{2}}{\rho_{1} \mathrm{v}_{1}{ }^{2} \mathrm{D}_{\mathrm{h}}} \mathrm{ds}
$$

Making the simplifying assumptions that the density and the friction coefficlent are approximately constant and applying conservation of mass,

$$
K_{\text {FRICTION }}=\lambda \mathrm{A}_{1}{ }^{2} \int_{0}^{s_{2}} \frac{\mathrm{ds}}{\mathrm{D}_{\mathrm{h}} \mathrm{~A}^{2}}
$$

which, for a conical diffuser, becomes

$$
\mathrm{K}_{\text {FRICTION }}=\frac{16 \mathrm{~A}_{1}{ }^{2} \lambda}{\pi^{2}} \int_{0}^{\mathrm{s}_{2}} \frac{\mathrm{ds}}{\mathrm{D}^{5}}
$$

and transforming variables from surface to centerline distances,

$$
K_{\text {FRICTION }}=\frac{16 A_{1}^{2} \lambda}{\pi^{2} \cos \theta} \int_{0}^{L} \frac{d x}{\left(D_{1}+2 x \tan \theta\right)^{5}}
$$

Completing this integration the friction loss becomes

$$
\mathrm{K}_{\text {FRICTION }}=\frac{\lambda}{8 \sin \theta}\left(1-\frac{1}{\mathrm{AR}^{2}}\right)
$$

The influence of the expansion term is given by

$$
\mathrm{K}=\mathrm{K}_{\text {EXPANSION }}+\mathrm{K}_{\text {FRICTION }}
$$

Thus, it may be rewritten:

$$
\begin{aligned}
& \mathrm{K}=\frac{\mathrm{K}_{\text {EXPANSION }}+\mathrm{K}_{\text {FRICTION }}}{\left(1-\frac{1}{\mathrm{AR}}\right)^{2}}\left(1-\frac{1}{\mathrm{AR}}\right)^{2} \\
& \mathrm{~K}=\left\{\mathrm{K}_{\mathrm{EXP}}+\left[\frac{\lambda}{8 \sin \theta} \frac{\frac{\mathrm{AR}^{2}-1}{A R^{2}}}{\left(\frac{A R-1}{A R}\right)^{2}}\right]\right\}\left(\frac{\mathrm{AR}-1}{\mathrm{AR}}\right)^{2} \\
& \mathrm{~K}=\left\{\mathrm{K}_{\mathrm{EXP}}+\left[\frac{\lambda}{8 \sin \theta}\left(\frac{\mathrm{AR}+1}{\mathrm{AR}-1}\right)\right]\right\}\left(\frac{\mathrm{AR}-1}{\mathrm{AR}}\right)^{2}
\end{aligned}
$$

$$
\begin{aligned}
& \mathrm{K}_{\mathrm{EXP}}=\frac{\mathrm{K}_{\text {EXPANSION }}}{\left(\frac{\mathrm{AR}-1}{\mathrm{AR}}\right)^{2}} \\
& \mathrm{~K}_{\mathrm{EXP}}=\frac{\mathrm{K}-\mathrm{K}_{\mathrm{FRICTION}}}{\left(\frac{\mathrm{AR}-1}{\mathrm{AR}}\right)^{2}}
\end{aligned}
$$

The expansion loss parameter curves shown in figure 4 were determined using the approximation

$$
\begin{aligned}
& \mathrm{K}_{\mathrm{EXP}}=\frac{\mathrm{K}-\mathrm{K}_{\text {FRICTION (CONICAL) }}}{\left(\frac{\mathrm{AR}-1}{\mathrm{AR}}\right)^{2}} \\
& \mathrm{~K}_{\mathrm{EXP}}=\frac{\mathrm{K}-\left[\frac{\lambda}{8 \sin \theta}\left(1-\frac{1}{\mathrm{AR}^{2}}\right)\right]}{\left(\frac{\mathrm{AR}-1}{\mathrm{AR}}\right)^{2}}
\end{aligned}
$$

and figure 5(a) of reference 9, which shows complete diffuser losses plotted as functions of equivalent cone angle and independent of area ratio for circular, square and rectangular, and two-dimensional diffusers. (This implies an assumption that the expansion part of the losses is dependent only on crosssectional shape, the extent to which the diffusion takes place in only one direction, and the equivalent cone angle.) Thus, the complete loss for diffusers is given as

$$
K=\left\{K_{E X P}+\left[\frac{\lambda}{8 \sin \theta}\left(\frac{A R+1}{A R-1}\right)\right]\right\}\left(\frac{A R-1}{A R}\right)^{2}
$$

using $K_{E X P}$ from figure 4.
Exit- The kinetic energy loss at an exit of a non-return wind tunnel was derived from basic compressibility relationships and with the assumptions that the exit flow static pressure is equal to the atmospheric pressure and that the exit velocity is uniform.

$$
\frac{p_{T}}{p}=\left[1-\left(\frac{\gamma-1}{2} M^{2}\right)\right]^{\frac{\gamma}{\gamma-1}}
$$

Rewriting, the local total pressure is

$$
p_{T}=p\left[1-\left(\frac{\gamma-1}{2} M^{2}\right)\right]^{\frac{\gamma}{\gamma-1}}
$$

Also, since $\Delta \mathrm{P}_{\mathrm{T}}=\mathrm{P}_{\mathrm{T}}-\mathrm{P}_{\mathrm{T}_{\mathrm{ATM}}}=\mathrm{P}_{\mathrm{T}}-\mathrm{p}$, the total loss parameter is

$$
K=\frac{\Delta p_{T}}{q}=\frac{p\left\{\left[1+\left(\frac{\gamma-1}{2} M^{2}\right)\right]^{\frac{\gamma}{\gamma-1}}-1\right\}}{\frac{1}{2} \gamma \mathrm{pM}^{2}}
$$

since

$$
\begin{equation*}
q=\frac{1}{2} \rho V^{2}=\frac{1}{2} \gamma p M^{2} \tag{ref.17,p.55}
\end{equation*}
$$

Simplifying, the exit loss becomes

$$
K=\frac{2}{\gamma M^{2}}\left\{\left[1+\left(\frac{\gamma-1}{2} M^{2}\right)\right]^{\frac{\gamma}{\gamma-1}}-1\right\}
$$

Flow straighteners: airfoil members (thick)- Thick flow straightener losses were assumed to be made up of two parts: contraction and subsequent diffusion:

$$
\mathrm{K}=\mathrm{K}_{\text {CONTRACTION }}+\mathrm{K}_{\text {DIFFUSION }}
$$

The contraction was estimated as being about 30 percent of the length of the straighteners:

$$
\begin{aligned}
& \mathrm{K}_{\text {CONTRACTION }}=\frac{0.32 \lambda(0.30 \mathrm{~L})}{\mathrm{D}_{\mathrm{h}}} \\
& \mathrm{~K}_{\text {CONTRACTION }}=\frac{0.096 \lambda \mathrm{~L}}{\mathrm{D}_{\mathrm{h}}}
\end{aligned}
$$

The diffusion portion was based on the aft 70 percent of the length and on the exit and minimum areas for the computation of the area ratio and equivalent cone angle. As for a vaned diffuser,

$$
\text { K DIFFUSION }=\left\{0.3+\left[0.006\left(2 \theta-21.5^{\circ}\right) \mathrm{u}\left(2 \theta-21.5^{\circ}\right)\right]\right\}\left(\frac{\mathrm{AR}-1}{\mathrm{AR}}\right)^{2}
$$

Hence the loss for thick flow straighteners becomes

$$
\mathrm{K}=0.096 \frac{\lambda \mathrm{~L}}{\mathrm{D}_{\mathrm{h}}}+\left\{0.3+\left[0.006\left(2 \theta-21.5^{\circ}\right) \mathrm{u}\left(2 \theta-21.5^{\circ}\right)\right]\right\}\left(\frac{\mathrm{AR}-1}{\mathrm{AR}}\right)^{2}
$$

Internal flow obstruction: drag item- The loss due to internal structure may be derived from the relationships governing power losses:
and $P_{D R A G}=D V \varepsilon=(1 / 2) \rho V^{3} S_{D} \varepsilon$, where $\varepsilon$ is the factor accounting for additional effects on downstream sections. Since $P_{\text {INPUT }}^{\text {DRAG }}$ $=P_{\text {DRAG }}$, the loss becomes

$$
K_{o}=\frac{q}{q_{o}} \frac{S}{A_{o}} C_{D} \varepsilon \frac{\rho_{F} V}{\rho_{o} V_{o}}
$$

and therefore

$$
\begin{aligned}
& K=C_{D} \frac{S}{A} \varepsilon \frac{\rho_{F}}{\rho} \frac{\rho V A}{\rho_{o} V_{o} A_{O}} \\
& K=C_{D} \frac{S}{A} \varepsilon \frac{\rho_{F}}{\rho}
\end{aligned}
$$

Since in general the flow density at the fans is unknown at the time a given section loss is calculated, and since for incompressible flow the density ratio is unity, the ratio of the densities at the fan and the local station was assumed as unity for the analysis. (If the user prefers not to make such an assumption, an approximation of the ratio may be made by way of a change in the downstream influence factor $\varepsilon$.) The loss due to a flow obstruction is

$$
K=C_{D} \frac{S}{A} \varepsilon
$$

Vaned diffusers- The expansion and friction losses for vaned diffusers were combined into one parameter which is reasonably independent of area ratio and is presented in figure 6. The loss curves shown were approximated by a two-segment, straight-line curve fit so that, for vaned diffusers

$$
K=K_{V}\left(\frac{A R-1}{A R}\right)^{2}
$$

and

$$
K=\left\{0.3+\left[0.006\left(2 \theta-21.5^{\circ}\right) u\left(2 \theta-21.5^{\circ}\right)\right]\right\}\left(\frac{A R-1}{A R}\right)^{2}
$$

where $u\left(2 \theta-21.5^{\circ}\right)$ is the unit step function.
Loss value transferred to reference location- The change of reference for loss values is defined as

$$
K_{0}=\frac{\Delta p_{T}}{q}\left(\frac{q}{q_{0}}\right)=K\left(\frac{q}{q_{0}}\right)
$$

Using the law of conservation of mass, this may be rewritten in terms of areas and Mach numbers:

$$
\begin{gathered}
\rho V A=\rho_{0} V_{0} A_{O} \\
\frac{q}{q_{0}}=\frac{\frac{1}{2} \rho V^{2}}{\frac{1}{2} \rho_{0} V_{o}^{2}}=\frac{A_{0} V}{A V_{O}} \\
\frac{q}{q_{O}}=\frac{A_{O}}{A} \frac{M}{M_{O}} \sqrt{\frac{1+\left(\frac{Y-1}{2} M_{o}^{2}\right)}{1+\left(\frac{Y-1}{2} M^{2}\right)}}
\end{gathered}
$$

and

$$
K_{o}=K\left[\frac{A_{0}}{A} \frac{M}{M_{0}} \sqrt{\frac{1+\left(\frac{Y-1}{2} M_{o}^{2}\right)}{1+\left(\frac{Y-1}{2} M^{2}\right)}}\right]
$$

Wall pressure differential- The pressure across a section of wall was determined from the exterior atmospheric pressure, internal static pressure, and cumulative pressure losses through the circuit. Since the wall pressure differential for a given section is $\Delta \mathrm{P}_{\mathrm{W}_{\mathbf{i}}}=\mathrm{p}_{\mathrm{T}_{\mathrm{ATM}}}-\mathrm{P}_{\mathrm{i}}$ and

$$
\mathrm{p}_{\mathrm{i}}=\frac{\mathrm{p}_{\mathrm{T}_{\mathrm{i}}}}{\left[1+\left(\frac{\gamma-1}{2} \mathrm{M}_{\mathrm{i}}{ }^{2}\right)\right]^{\frac{\gamma}{\gamma-1}}}
$$

and, using the test section as the reference location,

$$
\mathrm{P}_{\mathrm{T}_{i}}=\mathrm{P}_{\mathrm{T}_{\mathrm{SC}}}-\sum_{j=1}^{i} \mathrm{~K}_{\mathrm{o}_{j}}
$$

The wall pressure differential may be written as

Input power required- The power input to the flow required for operation of a wind tunnel circuit having predetermined losses was calculated from the pressure rise required at the fans, with the simplifying assumption that the static and total pressure rises across the fan are equal.

$$
\begin{aligned}
& \mathrm{P}_{\text {INPUT }}=\Delta \mathrm{P}_{\mathrm{F}} \mathrm{~A}_{\mathrm{F}} \mathrm{~V}_{\mathrm{F}} \\
& P_{\text {INPUT }}=\Delta p_{T_{F}} A_{F} V_{F} \frac{\rho_{F} \rho_{o} A_{O} V_{O}}{\rho_{F} \rho_{o} A_{o} V_{O}}
\end{aligned}
$$

Considering conservation of mass,

$$
\mathrm{P}_{\text {INPUT }}=\Delta \mathrm{p}_{\mathrm{T}} A_{\mathrm{o}} \mathrm{~V}_{\mathrm{o}} \frac{\rho_{\mathrm{O}}}{\rho_{\mathrm{F}}}
$$

Also,

$$
\Delta \mathrm{p}_{\mathrm{T}_{\mathrm{F}}}=\Delta \mathrm{p}_{\mathrm{T}}=\mathrm{q}_{\mathrm{o}} \sum_{\mathrm{i}=1}^{\mathrm{N}} \mathrm{~K}_{\mathrm{o}_{\mathrm{i}}}
$$

Thus,

$$
\begin{aligned}
& P_{\text {INPUT }}=\left(\sum_{i=1}^{N} K_{o_{i}}\right) \frac{1}{2} \rho_{o} A_{o} V_{o}^{3} \frac{\rho_{0}}{\rho_{F}} \\
& P_{\text {INPUT }}=\frac{\left(\sum_{i=1}^{N} K_{o_{i}}\right) \rho_{o}{ }^{2} A_{o} V_{o}^{3}}{2 \rho_{F}}
\end{aligned}
$$

## APPENDIX B

## NUMERICAL FUNCTION-APPROXIMATIONS

The formulas that follow resulted from curve-fitting and/or interpolation techniques applied to certain functions arising from the loss analysis.

## Corners

The corner loss parameters for corners with and without turning vanes are shown in figure 3. For a corner with vanes, using least-squares polynomial curve-fitting techniques, the turning vane loss function of figure 3(a) becomes, for $0^{\circ} \leq \emptyset \leq 30^{\circ}$ :

$$
\begin{align*}
\mathrm{K}_{\mathrm{TV}}= & 1.395066 \times 10^{-2}+5.672649 \times 10^{-4} \emptyset \\
& +7.081591 \times 10^{-5} \emptyset^{2}+1.394685 \times 10^{-6} \emptyset^{3} \\
& -4.885101 \times 10^{-8} \not \emptyset^{4} \tag{B1}
\end{align*}
$$

and for $30^{\circ}<\emptyset \leq 90^{\circ}$ :

$$
\begin{align*}
\mathrm{K}_{\mathrm{TV}}= & -1.605670 \times 10^{-1}+1.446753 \times 10^{-2} \phi \\
& -2.570748 \times 10^{-4} \phi^{2}+2.066207 \times 10^{-6} \phi^{3} \\
& -6.335764 \times 10^{-9} \phi^{4} \tag{B2}
\end{align*}
$$

For a corner without turning vanes the local loss function of figure 3(b) was found using a least-squares polynomial technique and is given by

$$
\begin{align*}
\mathrm{K}= & 4.313761 \times 10^{-5}-6.021515 \times 10^{-4} \phi \\
& +1.693778 \times 10^{-4} \phi^{2}-2.755078 \times 10^{-6} \phi^{3} \\
& +2.323170 \times 10^{-7} \phi^{4}-3.775568 \times 10^{-9} \phi^{5} \\
& +1.796817 \times 10^{-11} \phi^{6} \tag{B3}
\end{align*}
$$

For all the above equations, $\varnothing$ is the flow turning angle in degrees.

## Diffusers

The determination of the diffuser loss parameter is a complex operation. It depends on the cross-sectional shape and equivalent cone angle of the section. For a conical diffuser the expansion functions are, for $3^{\circ} \leq 2 \theta \leq 10^{\circ}$ :

$$
\left.\begin{array}{rl}
\mathrm{K}_{\text {EXP }}^{\text {Circular }}
\end{array}=1.70925 \times 10^{-1}-5.84932 \times 10^{-2}(2 \theta), ~(2 \theta)^{3}+1.34777 \times 10^{-4}(2 \theta)^{3}\right)
$$

for $0^{\circ}<2 \theta<3^{\circ}$ :

$$
\begin{equation*}
\mathrm{K}_{\mathrm{EXP}}^{\text {Circular }}=1.033395 \times 10^{-1}-1.19465 \times 10^{-2}(2 \theta) \tag{B5}
\end{equation*}
$$

and for $2 \theta>10^{\circ}$ :

$$
\begin{equation*}
\mathrm{K}_{\mathrm{EXP}}^{\text {Circular }}=-9.66135 \times 10^{-2}+2.336135 \times 10^{-2}(2 \theta) \tag{B6}
\end{equation*}
$$

For a square cross-section diffuser the expressions are, for $3^{\circ} \leq 2 \theta \leq 10^{\circ}$ :

$$
\left.\begin{array}{rl}
\mathrm{K}_{\text {EXP }}^{\text {Square }}
\end{array}=1.22156 \times 10^{-1}-2.29480 \times 10^{-2}(2 \theta), ~(2 \theta)^{2}-4.08644 \times 10^{-4}(2 \theta)^{3}\right)
$$

for $0^{\circ}<2 \theta<3^{\circ}$ :

$$
\begin{equation*}
\mathrm{K}_{\text {EXP }}^{\text {Square }}=9.62274 \times 10^{-2}-2.07582 \times 10^{-3}(2 \theta) \tag{B8}
\end{equation*}
$$

and for $2 \theta>10^{\circ}$ :

$$
\begin{equation*}
\mathrm{K}_{\text {EXP }}^{\text {Square }}=-1.321685 \times 10^{-1}+2.93315 \times 10^{-2}(2 \theta) \tag{B9}
\end{equation*}
$$

For a two-dimensional diffuser with a square upstream-end cross section the expansion loss functions are, for $3^{\circ} \leq 2 \theta \leq 9^{\circ}$ :

$$
\begin{align*}
& \mathrm{K}_{\mathrm{EXP}_{2 \mathrm{D}_{\text {Rectangular }}=}}=3.23334 \times 10^{-1}-5.82939 \times 10^{-2}(2 \theta) \\
&-4.97151 \times 10^{-2}(2 \theta)^{2}+1.99093 \times 10^{-2}(2 \theta)^{3} \\
&-1.98630 \times 10^{-3}(2 \theta)^{4}+2.06857 \times 10^{-5}(2 \theta)^{5} \\
&+3.81387 \times 10^{-6}(2 \theta)^{6} \tag{B10}
\end{align*}
$$

for $9^{\circ} \leq 2 \theta \leq 10^{\circ}$ :

$$
\begin{align*}
\mathrm{K}_{\mathrm{EXP}}^{2 \mathrm{D}}{ }_{\text {Rectangular }}= & 5.72853-1.21832(2 \theta) \\
& +7.08483 \times 10^{-2}(2 \theta)^{2} \tag{B11}
\end{align*}
$$

for $0^{\circ}<2 \theta<3^{\circ}$ :

$$
\begin{equation*}
\mathrm{K}_{\mathrm{EXP}_{2 \mathrm{D}}^{\text {Rectangular }}}=1.0 \times 10^{-1}-5.333333 \times 10^{-3}(2 \theta) \tag{B12}
\end{equation*}
$$

and for $2 \theta>10^{\circ}$ :

$$
\begin{equation*}
\mathrm{K}_{\mathrm{EXP}}^{2 \mathrm{D}_{\text {Rectangular }}}{ }=-1.36146+1.986460 \times 10^{-1} \tag{20}
\end{equation*}
$$

Since the expansion function for a two-dimensional diffuser with circular sides was not given (and is not defined), it was assumed for computational purposes that this value would be the same fraction of that for a twodimensional rectargular diffuser as the loss of a conical is of that for a three-dimensional square diffuser:

$$
\left.\mathrm{KEXP}_{2 \mathrm{D}_{\text {Circular }}}=\mathrm{K}_{\mathrm{EXP}_{2 \mathrm{D}_{\text {Rectangular }}}\left(\frac{\mathrm{K}_{\text {EXP }} \text { Circular }}{}\right.}^{\mathrm{K}_{\mathrm{EXP}}^{\text {Square }}} \text { }\right)
$$

For cross-section shapes somewhere between rectangular and circular, such as flat oval (flat ceiling and floor with semi-circular sidewalls), or for diffusers which have one end rectangular and one end either circular or flat oval, a loss value between that for circular and rectangular may be more appropriate; thus,

$$
\mathrm{K}_{\text {EXP }}^{2 \mathrm{D}_{\text {Average }}} \boldsymbol{=} \frac{\mathrm{K}_{\mathrm{EXP}}^{2 \mathrm{D}_{\text {Rectangular }}}{ }+\mathrm{K}_{\text {EXP }}^{2 \mathrm{D}_{\text {Circular }}}}{}
$$

and

The extent to which a diffuser is planar in nature was computed from the ratio of the changes in size of the two characteristic dimensions from end to end:

$$
\delta_{s}=\text { smaller of } \frac{h_{2}-h_{1}}{w_{2}-w_{1}} \text { or } \frac{w_{2}-w_{1}}{h_{2}-h_{1}}
$$

or if the ratio is negative,

$$
\delta_{s} \equiv 0
$$

Then, based on the geometries of each end, the basic loss constant, $K_{E X P}{ }_{\text {Basic }}$, may be selected from $\mathrm{K}_{\mathrm{EXP}}$ Circular, $\mathrm{K}_{\mathrm{EXP}}^{3 \mathrm{D}_{\text {Average }}}$ or $\mathrm{K}_{\mathrm{EXP}}$ Square and the additional loss fact. $K_{E X P}$ Additional, may be selected from the corresponding
$\mathrm{K}_{\mathrm{EXP}}^{2 \mathrm{D}}$ Circular , $\mathrm{K}_{\mathrm{EXP}}^{2 \mathrm{D}_{\text {Average }}}$ or $\mathrm{K}_{\mathrm{EXP}}^{2 \mathrm{D}_{\text {Rectangular }} \text {. Finally, the applicable }}$ diffuser expansion loss coefficient is given by

$$
\begin{equation*}
\mathrm{K}_{\mathrm{EXP}}=\mathrm{K}_{\mathrm{EXP}}^{\text {Basic }} \text { }+\left(1-\delta_{\mathrm{s}}\right)\left(\mathrm{K}_{\mathrm{EXP}}^{\text {Additional }} 1-\mathrm{K}_{\mathrm{EXP}}^{\text {Basic }} \text { }\right) \tag{B14}
\end{equation*}
$$

## Mesh Screen

The mesh screen Reynolds number sensitivity factor plotted in figure 5 can be expressed in functional form as, for $0 \leq \mathrm{RN}<400$ :

$$
\begin{equation*}
\mathrm{K}_{\mathrm{RN}}=\frac{78.5\left(1-\frac{\mathrm{RN}}{354}\right)}{100}+1.01 \tag{B15}
\end{equation*}
$$

and for $\mathrm{RN} \geq 400$ :

$$
\mathrm{K}_{\mathrm{RN}} \equiv 1.0
$$

## Vaned Diffusers

The vaned diffuser loss coefficient functions plotted in figure 6 were approximated by two line segments; for $2 \theta<21.5^{\circ}$ :

$$
K_{v}=0.3
$$

and for $21.5^{\circ} \leq 2 \theta \leq 90^{\circ}$ :

$$
K_{v}=0.3+\left[0.006\left(2 \theta-21.5^{\circ}\right)\right]
$$

Thus, over the entire range of equivalent cone angles of interest,

$$
\begin{equation*}
\mathrm{K}_{\mathrm{V}}=0.3+\left[0.006\left(2 \theta-21.5^{\circ}\right) \mathrm{u}\left(2 \theta-21.5^{\circ}\right)\right] \tag{B16}
\end{equation*}
$$

where $u\left(2 \theta-21.5^{\circ}\right)$ is the unit step function.

APPENDIX C

## COMPUTER PROGRAM FØRTRAN CODES

The following pages contain the FøRTRAN codes developed to implement the wind tunnel performance analysis techniques presented in this report.

The Notation section explains the variable names used in the program. (Note that in the notation sections, as throughout this report, all letter 0's occurring in F $\emptyset$ RTRAN names are shown with slashes, as $\emptyset$; all number zeros are shown unslashed.) This notation section is similar to that for engineering symbols presented in the main body of the report, but this section was changed in two respects. First, it was rearranged alphabetically by FøRTRAN variable name. Second, it was expanded to include many variable names which were not used elsewhere and which have significance only in the context of the computer program. The "titles" shown in parentheses in the first column of this notation section are column heading titles which appear on the program output pages.

Immediately following the Notation are the listings of the six actual FøRTRAN program codes: the main program (PERFøRM) and the five subroutines
 titled and numbered for clarity. The last seven columns of each line on each page contain a two-letter program routine name abbreviation and a line sequence number (in ten-count increments). Thus, the user can know at a glance to which routine (and where within that routine) any given line or instruction belongs. Each instruction line in the program is uniquely identified.

| FøRTRAN name and/or (title) | $\begin{gathered} \text { Engineering } \\ \text { symbol } \end{gathered}$ | Description |
| :---: | :---: | :---: |
| A | A | ```cross-sectional area of local section, m``` |
| AII |  | cross-sectional area of individual duct at upstream end, $m^{2}\left(\mathrm{ft}^{2}\right)$ |
| AI2 |  | ```cross-sectional area of individual duct at downstream end, m}\mp@subsup{}{}{2}(f\mp@subsup{t}{}{2}``` |
| AL | AFLOW | ```cross-sectional area of local flow, m}\mp@subsup{}{}{2 (ft')``` |
| AMACH | M | local Mach number |
| AMACHI <br> (MACH1) |  | Mach number at section upstream end |
| AMACH2 <br> (MACH2) |  | Mach number at section downstream end |
| $\begin{aligned} & \mathrm{AR} \\ & (\mathrm{AR}, \mathrm{CR}) \end{aligned}$ | AR | ratio of cross-sectional areas at upstream and downstream ends of section |
| ASL |  | speed of sound in moving flow at local section, $\mathrm{m} / \mathrm{sec}(\mathrm{ft} / \mathrm{sec})$ |
| ASTAR | $A_{*}$ | cross-sectional area for sonic flow at specified flow conditions, $\mathrm{m}^{2}\left(\mathrm{ft}^{2}\right)$ |
| ASO | $\mathrm{a}_{0}$ | speed of sound in moving flow at upstream end of test section, $m / s e c$ ( $f t / s e c$ ) |
| AT | ${ }^{\text {a }}$ T | speed of sound in still gas, computed at total (stagnation) conditions, $\mathrm{m} / \mathrm{sec}$ (ft/sec) |
| AVGPWR |  | average power consumed by each drive fan at specified conditions: PWRøP/ENFAN, W (hp) |
| AO | $\mathrm{A}_{0}$ | cross-sectional flow area of test section at upstream end, $\mathrm{m}^{2}$ ( $\mathrm{ft}^{2}$ ) |
| A1 <br> (AREA1) | $\mathrm{A}_{1}$ | cross-sectional flow area of section at upstream end, $m^{2}\left(f t^{2}\right)$ |


| FøRTRAN name and/or (title) | $\begin{gathered} \text { Engineering } \\ \text { symbol } \end{gathered}$ | Description |
| :---: | :---: | :---: |
| $\begin{aligned} & \mathrm{A} 1 \emptyset \mathrm{AO} \\ & (\mathrm{~A} 1 / \mathrm{AO}) \end{aligned}$ |  | ratio of local section upstream area to test section area, $\mathrm{m}^{2}\left(\mathrm{ft}^{2}\right)$ |
| $\begin{aligned} & \text { A2 } \\ & \text { (AREA2) } \end{aligned}$ | $\mathrm{A}_{2}$ | cross-sectional flow area of section at downstream end, $m^{2}\left(f t^{2}\right)$ |
| $\begin{aligned} & \mathrm{A} 2 \emptyset \mathrm{AO} \\ & (\mathrm{~A} 2 / \mathrm{AO}) \end{aligned}$ |  | ratio of local section downstream area to test section area, $\mathrm{m}^{2}\left(\mathrm{ft}^{2}\right)$ |
| BLKAGE |  | blockage to flow in local section (at upstream end for all applicable sections except fan contraction, for which it is at downstream end), fraction of local area |
| (BLKGE) |  | blockage to flow in local section (at upstream end for all applicable sections except fan contraction, for which it is at downstream end), percent of local area |
| CD | $\mathrm{C}_{\text {D }}$ | drag coefficient of flow obstruction, $\frac{\mathrm{drag}}{\mathrm{q} S}$ |
| CHøRD | $\mathrm{c}_{\mathrm{v}}$ | chord of turning vanes, m (ft) |
| D | D | diameter of circular duct, m (ft) |
| DATA |  | data array of master, section, and termination card floating-point inputs |
| DELP | $\Delta \mathrm{p}_{\mathrm{W}_{\mathrm{i}}}$ | local pressure difference across wind tunnel wall, $\mathrm{N} / \mathrm{m}^{2}\left(\mathrm{lb} / \mathrm{ft}^{2}\right)$ |
| (D EPS) | $\Delta \varepsilon$ | increment of flow-obstruction downstream influence factor greater than unity: $\varepsilon-1$, (greater than or equal to zero) |
| DFAN |  | drive fan diameter, m (ft) |
| DH | $\mathrm{D}_{\mathrm{h}}$ | $\begin{aligned} & \text { hydraulic diameter: } \\ & \frac{4 \times(\text { cross-sectional area })}{\text { perimeter }}, \mathrm{m}(\mathrm{ft}) \end{aligned}$ |
| DHL |  | hydraulic diameter of single cell in flow straightener, m (ft) |


| FøRTRAN name and/or (title) | Engineering symbol | Description |
| :---: | :---: | :---: |
| DHUB |  | ```diameter of drive fan hub and/or spinner, m (ft)``` |
| DHO |  | ```hydraulic diameter of test section, m (ft)``` |
| DH1 |  | hydraulic diameter of upstream end of local section, m (ft) |
| DH2 |  | hydraulic diameter of downstream end of local section, m (ft) |
| DMESH |  | ```diameter of mesh element in woven-mesh screen, m (ft)``` |
| D1 |  | diameter of upstream end of circular section, m (ft) |
| D2 |  | diameter of downstream end of circular section, m (ft) |
| $\begin{aligned} & \text { EK } \\ & \text { (DP/QL) } \end{aligned}$ | K | ```local total pressure loss of section: \Delta\mp@subsup{p}{T}{}``` |
| EKADD | $\mathrm{K}_{\text {EXP }}{ }_{\text {Additional }}$ | additional diffuser expansion loss factor due to more diffusion in one plane than in another (i.e., partially twodimensional diffusion) |
| EKBASE | $\mathrm{K}_{\text {EXP }}{ }_{\text {Basic }}$ | basic diffuser expansion loss factor for purely three-dimensional diffusion |
| EKC | $\mathrm{K}_{\text {EXP }}{ }_{\text {Circular }}$ | expansion loss value for conical diffusers |
| EKCNTR | $\mathrm{K}_{\text {CONTRACTION }}$ | local total pressure loss from contracting portion of thick-airfoil flow straighteners |
| EKCSAV | $\mathrm{K}_{\mathrm{EXP}}^{3{ }_{3 D}}{ }_{\text {Average }}$ | estimated expansion loss coefficient for three-dimensional, combination circular and square cross-section diffuser |
| EKD | K DIFFUSION | local total pressure loss from diffusing portion of multi-loss-type sections |
| $\begin{aligned} & \text { EKEXP } \\ & \text { (KEXP) } \end{aligned}$ |  | net expansion loss coefficient for diffusers |


| FØRTRAN name and/or (title) | Engineering symbols |
| :---: | :---: |
| EKMESH <br> (KMESH) | $\mathrm{K}_{\text {MESH }}$ |
| EKS | $\mathrm{K}_{\text {EXP }}$ Square |
| EKSTRT |  |
| EKTE |  |
| $\begin{aligned} & \text { EKTE90 } \\ & \text { (KT 90) } \end{aligned}$ |  |
| EKTV | $\mathrm{K}_{\mathrm{TV}}$ |
| $\begin{aligned} & \text { EKTV90 } \\ & \text { (KT 90) } \end{aligned}$ | $\mathrm{K}_{\mathrm{TV}}^{90}$ |
| EKV | $\mathrm{K}_{\mathrm{v}}$ |
| $\begin{aligned} & \text { EKO } \\ & \text { (DP/QO) } \end{aligned}$ | $\mathrm{K}_{0}$ |

EKI

EK2

EK2DC

EK2DCS
$\mathrm{K}_{\mathrm{EXP}}^{2 \mathrm{D}_{\text {Circular }}}{ }$
$\mathrm{K}_{\mathrm{EXP}}^{2 \mathrm{D}_{\text {Average }}}{ }$
mesh screen-type loss constant
expansion loss value for threedimensional expansion in square crosssection diffusers
local total pressure loss coefficient due to strut drag in fan section
local total pressure loss parameter for corners without turning vanes
vaneless-corner loss parameter for given corner at a $90^{\circ}$ turn
turning vane loss coefficient
turning vane loss parameter for given vanes at a $90^{\circ}$ turn
local total pressure loss coefficient for vaned diffusers
section total pressure loss referred to test section conditions: $\frac{\Delta \mathrm{P}_{\mathrm{T}}}{\mathrm{q}_{\mathrm{o}}}$
local total pressure loss coefficient due to diffusion and vanes in a diffusing corner
local total pressure loss coefficient due to rotational flow in a diffusiing corner
estimated expansion loss coefficient for hypothetical, two-dimensional diffusion with circular sides:
$\mathrm{K}_{\text {EXP }_{2 D_{\text {Rectangular }}}}\left(\frac{\mathrm{K}_{\text {EXP }} \text { Circular }}{}\right)$
estimated expansion loss coefficient for two-dimensional diffuser with crosssection shape of some square/circular hybrid

| FøRTRAN name and/or (title) | Engineering symbol | Description |
| :---: | :---: | :---: |
| EK2DR | $\mathrm{K}_{\mathrm{EXP}_{2} \mathrm{D}_{\text {Rectangular }}}$ | expansion loss coefficient for twodimensional rectangular cross-section diffusers |
| EL <br> (L) | L | centerline length of section, m (ft) |
| ELC |  | length of contracting portion of thickairfoil flow straighteners, m (ft) |
| ELD |  | length of diffusing portion of thickairfoil flow straighteners, m (ft) |
| $\begin{aligned} & \text { EL } \varnothing \mathrm{DH} \\ & \text { (L/DH) } \end{aligned}$ |  | length-to-hydraulic-diameter ratio of flow straightener cell |
| EMDATA |  | data array containing master-card floating-point inputs |
| EMF |  | Mach number at the fan section |
| EMU | $\mu$ | flow viscosity, $\mathrm{N} \mathrm{sec} / \mathrm{m}^{2}\left(1 \mathrm{~b} \mathrm{sec} / \mathrm{ft}^{2}\right)$ |
| EMUSTD | $\mu_{\text {std }}$ | ```standard-day value of viscosity, N sec/m``` |
| EMUT | ${ }^{\text {T }}$ | reference viscosity at a known temperature, computed for a still gas (stagnation conditions), $\mathrm{N} \sec / \mathrm{m}^{2}$ ( $\mathrm{lb} \mathrm{sec} / \mathrm{ft}^{2}$ ) |
| EMWRIT |  | master card output array containing data and/or annotation(s) |
| EMO | M | Mach number at upstream end of test section |
| ENDATA |  | data array containing section-card floating-point input |
| ENDUCT |  | number of ducts in multiple-duct sections |
| ENFAN |  | number of fans in fan drive section |
| ENITEM |  | number of drag or blockage items in each local duct |
| ENU | $v$ | kinematic viscosity of gas, $\mathrm{m}^{2} / \mathrm{sec}$ ( $\mathrm{ft}{ }^{2} / \mathrm{sec}$ ) |


| FøRTRAN name and/or (title) | Engineering system | Description |
| :---: | :---: | :---: |
| ENWRIT |  | section-card output array containing data and/or annotation(s) |
| EPS | $\varepsilon$ | flow-obstruction downstream influence factor (greater than or equal to unity) |
| ER | ER | energy ratio: ratio of energy of flow at test section to the output energy of the fans |
| ETAFAN <br> (ETA) | ${ }^{n}$ F | fan aerodynamic efficiency, percent |
| ETWRIT |  | case termination-card output array containing termination request de-codings |
| FAC |  | function defining the area of sections with circular cross sections |
| FACT |  | scaling factor for plot size |
| FAFø |  | function defining the area of sections with flat-oval cross sections (flat floor and ceiling, semi-circular walls) |
| FAR |  | function defining the area of sections with rectangular cross sections |
| FDHC |  | function defining the hydraulic diameter of sections with circular cross sections |
| FDHFD |  | function defining the hydraulic diameter of sections with flat-oval cross sections |
| FDHR |  | function defining the hydraulic diameter of sections with rectangular cross sections |
| FEKC |  | function defining the diffuser expansion loss for three-dimensional, circular cross-section diffusers |
| FEKCH |  | function defining the diffuser expansion loss for three-dimensional, circular cross-section diffusers at high diffusion angles (TH2 > $10^{\circ}$ ) |

FøRTRAN name and/or (title) FEKCS

FEKS

FEKSH

FEKSS

FEKO

FEK2DL

FEK2DU

FKTE

FKTV1

FKTV2
$f(\phi)$
$\mathrm{f}(\phi)$

FTH
function defining the diffuser expansion loss for three-dimensional, circular cross-section diffusers at small diffusion angles (TH2 < $3^{\circ}$ )
function defining the diffuser expansion loss for three-dimensional, square cross-section diffusers
function defining the diffuser expansion loss for three-dimensional, crosssection diffusers at high diffusion angles (TH2 > $10^{\circ}$ )
function defining the diffuser expansion loss for three-dimensional, square cross-section diffusers at small diffusion angles (TH2 < $3^{\circ}$ )
function defining the change-of-reference station for total pressure losses from local section to test section
function defining "two-dimensional" (rectangular) diffuser expansion loss for low diffuser angle range (TH2 < $9^{\circ}$ )
function defining "two-dimensional" (rectangular) diffuser expansion loss for high diffuser angle range (TH2 $29^{\circ}$ )
function defining corner turning loss parameter EKTE for corners without turning vanes (based on a value of EKTE $=1.80$ at $\mathrm{PHI}=90^{\circ}$ )
function defining turning vane loss parameter EKTV (based on a value of EKTV $=0.15$ at PHI $=90^{\circ}$ ) for lower turning angle range ( $\mathrm{PHI} \leq 30^{\circ}$ )
function defining turning vane loss parameter EKTV (based on a value of EKTV $=0.15$ at PHI $=90^{\circ}$ ) for upper turning angle range ( $30^{\circ}<\mathrm{PHI} \leq 90^{\circ}$ )
function converting diffuser equivalent cone angle, TH2, in degrees to halfangle, TH , in radians

| FøRTRAN name and/or (title) | Engineering symbol | Description |
| :---: | :---: | :---: |
| FTH2 |  | function defining diffuser equivalent cone angle, TH2 |
| G | $\gamma$ | specific heat ratio of gas |
| H1 | $\mathrm{h}_{1}$ | height at the upstream end of a noncircular section |
| H2 | $\mathrm{h}_{2}$ | height at the downstream end of a noncircular section |
| IFLAG |  | parameter indicating the sequence number assigned to the fan section |
| IPL $\emptyset$ T |  | decision parameter for selecting which (if any) plots are to be plotted |
| IPRINT |  | decision parameter for requesting or omitting output of summary characteristics page |
| ISEC |  | section type-description code |
| ISEQ |  | input section sequence number |
| ISHAP1 |  | ```section upstream-end cross-sectional shape code``` |
| ISHAP2 |  | ```section downstream-end cross-sectional shape code``` |
| Ititla |  | assumed case-title array in the event the title card is omitted |
| ITITLE |  | input case-title array |
| ITUNNL |  | wind tunnel circuit-type code |
| ITYPE |  | code for type of output format required for printing section information |
| IU |  | units-of-measure type code |
| LINEMX |  | maximum number of output lines per page |
| LMTSEC |  | limit for maximum number of sections in any given case |


| FøRTRAN name and/or (title) | Engineering system | Description |
| :---: | :---: | :---: |
| MCHECK |  | ```master-card input-requirement checking code array``` |
| MDATA |  | master-card integer input data array |
| MFøRMT |  | master-card output format array |
| MWRITE |  | master-card output array containing data and/or annotation(s) |
| N | N | section assigned sequence number for order of occurrence in circuit |
| NCHECK |  | ```section-card input-requirement checking code array``` |
| NDATA |  | section-card integer input data array |
| NFøRMT |  | section-card output format array |
| NN |  | section type number for printing proper section title |
| NWRITE |  | section-card output array containing data and/or annotation(s) |
| P |  | input tunnel total (stagnation) pressure, standard atmospheres |
| PA |  | input atmospheric (barometric) pressure, standard atmospheres |
| PATM <br> (P ATM) | ${ }^{\mathrm{P}} \mathrm{T}_{\text {ATM }}$ | $\begin{aligned} & \text { atmospheric (barometric) pressure, } \\ & \mathrm{N} / \mathrm{m}^{2}\left(1 \mathrm{~b} / \mathrm{ft}^{2}\right) \end{aligned}$ |
| PHI | $\phi$ | corner flow turning angle, deg |
| PI | $\pi$ | ratio of the area of a circle to the square of its radius |
| PRSTY |  | porosity of certain non-solid flow obstructions: AL/A |
| PT | $\mathrm{p}_{\mathrm{T}}$ | ```tunnel total (stagnation) pressure, N/m (1b/ft }\mp@subsup{}{}{2}\mathrm{ )``` |
| PWRI |  | decision parameter for requesting or omitting the matching of power consumption with given input value |


| FØRTRAN name and/or (title) | Engineering system | Description |
| :---: | :---: | :---: |
| PWRIP |  | power required to be input to flow in order to drive wind tunnel at specified speed, W (hp) |
| PWRMCH |  | total power value for which the maximum test section velocity is to be determined (if requested), $W$ (hp) |
| PWR P $^{\text {P }}$ | $\mathrm{P}_{\text {REQUIRED }}$ | total fan motor output power required to drive wind tunnel at specified speed, W (hp) |
| Q0 | $\mathrm{q}_{0}$ | $\begin{aligned} & \text { test section upstream-end dynamic } \\ & \text { pressure: } \frac{\rho_{0} V_{0}{ }^{2}}{2}, N / m^{2}\left(1 b / f t^{2}\right) \end{aligned}$ |
| R | R | gas constant, $\mathrm{m}^{2} / \mathrm{sec}^{2}{ }^{\circ} \mathrm{K}\left(\mathrm{ft}^{2} / \mathrm{sec}^{2}{ }^{\circ} \mathrm{R}\right)$ |
| RHøS | $\rho$ | $\begin{aligned} & \text { local static density, } N \sec ^{2} / m^{4} \\ & \left(1 b \sec ^{2} / f t^{4}\right) \end{aligned}$ |
| RHøSF | $\rho_{\mathrm{F}}$ | static density at the fans, $N \sec ^{2} / \mathrm{m}^{4}$ (lb $\sec ^{2} / f t^{4}$ ) |
| RHøS0 | $\rho_{0}$ | static density at upstream end of test section, $N \sec ^{2} / \mathrm{m}^{4}\left(1 b \sec ^{2} / \mathrm{ft}^{4}\right)$ |
| RHø ${ }^{\text {P }}$ | ${ }^{\rho}$ T | density computed for total (stagnation) conditions, $N \sec ^{2} / \mathrm{m}^{4}\left(1 \mathrm{~b} \mathrm{sec}{ }^{2} / \mathrm{ft}^{4}\right)$ |
| RN | RN | Reyno1ds number: $\frac{\rho V \ell}{\mu}$ |
| RNREF | $\mathrm{RN}_{\text {REF }}$ | reference Reynolds number at which turning vane $90^{\circ}$-loss constant, EKTV90, was determined |
| RNV |  | Reynolds number for turning vanes based on vane chord: $\frac{\rho V c_{V}}{\mu}$ |
| RUFNES | $\Delta$ | ```surface roughness in honeycomb cells, m (ft)``` |
| (RUFNES) |  | $\begin{aligned} & \text { surface roughness in honeycomb cells, } \\ & 10^{-6} \mathrm{~m}\left(10^{-6} \mathrm{ft}\right) \end{aligned}$ |
| SEKO |  | section total pressure loss array (referenced to test section conditions) used in summary calculations |


| FøRTRAN name and/or (title) | Engineering symbol | Description |
| :---: | :---: | :---: |
| SEL |  | section centerline length array used in summary calculations, m (ft) |
| SERR $\emptyset$ R |  | section input error occurrence code |
| Slamda |  | friction coefficient for smooth pipes |
| SLmDAE |  | calculated friction coefficient in test section at the requested power-matching condition |
| SLMDA1 |  | friction coefficient at section upstream end |
| SLMDA2 |  | friction coefficient at section downstream end |
| SLR | $\delta_{s}$ | diffuser side length ratio: ratio of change in height to change in width from upstream to downstream end, or its inverse, whichever is less than or equal to unity |
| SMACH |  | section downstream-end Mach number array used in summary calculations |
| SøA <br> (S/AL) |  | ratio of flow-obstruction drag area to local flow area |
| SSUMEL |  | summation array of total centerline length from start of circuit to end of local section |
| SSUMKO |  | summation array of total pressure losses from start of circuit to end of local section |
| SUMEKO | $\sum_{i=1}^{N} K_{o_{i}}$ | summation of all section total pressure losses referenced to test section conditions |
| SUMEL | $\sum_{i=1}^{N} L_{i}$ | ```summation of all section centerline lengths (total circuit flow length), m (ft)``` |
| T |  | ```tunnel total (stagnation) temperature, * C ( }\mp@subsup{}{}{\circ}\textrm{F}\mathrm{ )``` |
| TH | $\theta$ | diffuser half-angle, rad |


| FøRTRAN name and/or (title) | Engineering symbol | Description |
| :---: | :---: | :---: |
| $\begin{aligned} & \text { TH2 } \\ & (2 \text { THETA) } \end{aligned}$ | $2 \theta$ | diffuser equivalent cone angle, deg |
| TLIST |  | case-fatal error occurrence code |
| TLISTI |  | decision parameter for requesting or omitting tabulation of input data |
| TRETRN |  | decision parameter for requesting return for additional case or final termination |
| TSBLKG |  | test section blockage used for computation of basic test section conditions, percent of test section cross-sectional area |
| TT | $\mathrm{T}_{\mathrm{T}}$ | ```tunnel total (stagnation) temperature, 0}\textrm{K}(\mp@subsup{}{}{\circ}\textrm{R}``` |
| V | V | local flow velocity, m/sec (ft/sec) |
| Voc |  | calculated test section velocity at adjusted power level, $\mathrm{m} / \mathrm{sec}$ (ft/sec) |
| VOK |  | test section flow velocity at input conditions, knots |
| V1 |  | ```section upstream-end flow velocity, m/sec (ft/sec)``` |
| V2 |  | ```section downstream-end flow velocity, m/sec (ft/sec)``` |
| W1 | ${ }^{\mathbf{w}} 1$ | width of upstream end of non-circular section, m (ft) |
| W2 | $\mathrm{w}_{2}$ | width of downstream end of non-circular section, m (ft) |



| C............ <br> Cobepressure hoss repergneestation iranger function | $\begin{aligned} & \rho M \\ & \rho M \end{aligned}$ | $\begin{array}{r} 510 \\ 520 \end{array}$ |
| :---: | :---: | :---: |
| C.e. PKKOREM | PM | 530 |
|  | PM | 540 |
| 13 SORT (EMOSO/(1.t(0-1.)/2.\#AMACH**2)) | PM | 550 |
|  | P1 | 500 |
| C.P. 6088 PARAMETER CURVEOFTT PUNCTIONS | PM | 570 |
|  | PM | 880 |
|  | PM | 590 |
|  |  | 600 |
|  | OM | 610 620 |
|  | PM | 630 |
|  | PM | 600 |
|  | PM | 650 |
|  | PM | 660 |
|  | PM | 670 |
|  | PM | 680 |
| PEKCE(TH2) - $10333950.119465 E \\|$ IH2 | PM | 690 |
| PEKgs(TH2) - 962274E-10,207382Ew2apHz | PM | 700 |
| FEK208(THE) dE.3333333E-2,TH2 | PM | 710 |
|  | PM | 720 |
| FEKSH(TH2) - 1321685*.293315Ea1*TH2 | PM | 730 |
|  | PM | 140 |
|  | PM | 750 |
|  | PM | 760 |
| $2$ | PM | 770 |
|  | PM | 780 |
|  | PM | 790 |
|  | PM | 800 |
|  | PM | 810 |
|  | PM | 820 |
| C.E.COMPILE-TIME ARMMETER DEFINITION8 | PM | 830 |
|  | PM | 840 |
| C.e.es | PM | 850 |
| EAR PIXED PARAMETER DEPINITIDNA | PM | 060 |
| C. | PM | 870 |
| Pi matapanelal | PM | 880 |
| PLOTON 0.0 | PM | 890 |
| C-SORO- | PM | 000 |
|  | PM | 910 |
|  | PM | 020 |
|  | PM | 930 |
| GNEMX - | PM | 920 |
| LMTEEC 3n | PM | 950 |
|  | PM | 260. |
| C. THERE IS NO RETURN PO PHE PREIOUS INsTRUCTIONS. | F | 970 |
| C.O. |  | 280 |
|  |  | 990 |
|  |  | 1000 |



| C.OAO- InPERNATIONAL SYSTEM OF UNITS (SI) | $\begin{array}{ll} \text { PM } & 1510 \\ \text { PM } & 1520 \end{array}$ |
| :---: | :---: |
| C. | PM 1530 |
|  | RM 1540. |
| PT E P 101325. | PM 1550 |
| PAIM PA, 101325. | PM 1560 |
| R = 280.79 | PM 1570 |
| TT $=$ It 273,15 | PM 1580 |
| EMUT EMUSTD* (TT/288,0)**,76 | PM 1590 |
| PWRMCE DAPA $71+1$-E6 | PM 1600 |
| 6010104 | PM 1610 |
|  | PM 1620 |
| C.: U.3. CUSTOMARY UNIT8 | PM 1630 |
|  | PM 1640 |
| 103 EMUSTD a 3.719 E -7 | PM 1650 |
| PT Pe2116.217 | PM 16.60 |
| -ATM - ${ }^{\text {PA*2116.217 }}$ | PM 8670 |
| 0.17150 | PM 1680 |
| TT- 74459.6 | PM 1690 |
| EmuT E EmUSTO*(TT1518.4)**.76 | PM 1700 |
| PWAMCH DATA(7)*1,E3 | PM 1710 |
| Crencerene | PM 1720 |
| C.E.GENERALEFORM DIMENEIONAL PARAMETERS | PM 1730 |
| Cees | PM 1740 |
| 104 AT SORT(GAR*TY) | PM 1790 |
| RHAT PT/(R⿴TI) | PM 1760 |
|  | PM 1770 PM 1780 |
| C... | PM 1790 |
| YO OAPA(b) | PM 1800 |
| 1n5 IF (IU eEO, iJ Vok Vowis9438 | PM 1810 |
| IF IU EGA 21 VOK VOL 59248 | PM 1820 |
| EMO VOAAT | PM 1830 |
|  | PM 1840 |
| EM VO/ASO | PM 1850 |
|  | PM 1860 |
| EMO EM | PM 1870 |
| 60 ro 106 | PM 1880 |
| 107 EMO EM | PM 1890 |
| C-cenerenes | Pm 1900 |
| C.... MACHANUMBEq=DEPENDENT PARAMETERS | PM 1910 |
|  | PM 1920 |
|  | PM 1930 |
|  | PM 1940 |
| QO RHOSO. VOME2/2. | PM 1950 |
| C-apesperse | PM 1060 |
| C.E.ETEST SECTION THROAT SLZE AND TMROAT-AREA DEPENDENT PARAMETERS | PM 1970 |
| C.O. | PM 1980 |
| T886KG D DAFA(5) | PM 1990 |
|  | PM 2000 |






| A1 : PI*(DFAN**2-DHUB**2)/4.*(1. $=$ BLKAGE/800.) *ENDUET AII A1/ENDUCI | $\begin{aligned} & \text { PM } 4010 \\ & \text { PH } 4020 . \end{aligned}$ |
| :---: | :---: |
|  | PM 4030 |
|  | PM 4040 |
| C. 0 | PM 4050 |
| C2is Al010 - A1/40 | PM 4060 |
| A2040 AR/AO | PM 4070 |
| AR AR/A1 | PM 4080 |
|  | PM 4090 |
| Call speEdialimachav) | PM.4100 |
| CALL FRICTN(OHL, A1, AMACH, SLAMDA) | PM 4110 |
| AMACHI AmACH | PM. 4120 |
| V1.V | PM 4130 |
| SLMOAI - SLAMOA | PM 4140 |
| CALL SPEED(AZ,AMACH, V) | PM 4150 |
| CALL FRIGINCDH2,A2,AMACM, SIAMDAI | RM 4160 |
| AMACHE AMACH | PM 4170 |
| SMACHPN) S AMACH2 | PM 4180 |
|  | PM 4190 |
| SLMDAZ SLAMDA | PM 4200 |
| IF IISEC NE 3 , ANO, ISEC NE, 4 AND, ISEC NE. 40 , AND. | PM 4210 |
|  | PM 4220 |
| 2 , AND. ISEC.NE, 94, GO PO 224 | PM 4230 |
|  | PM.4240 |
| C.O.DEFINİION OF DIPFUSER.ONLY PARAMETERS | PM 4250 |
|  | PM 4200 |
| TH2 FTH2 (AZ,A1,EL) | PM 4270 |
|  | PM 4280. |
| TH FTH(TH2) | PM 4290 |
| EKEXP = DATA(12) | PM 4300 |
| IP (EKEXP, GT. 1.E06) G0 T0 224 | PM 4310 |
| Cesens | PM 4320 |
| C.: DEFAULT-CAUSED DEPERMINATIUN OF DIFFUSER EXPANSION LO88 | PM 4330 |
| CRS PARAYEIER | DM 4340 |
| C. ${ }^{\text {c }}$ | Pm 4350 |
| C.EKC FEKG(TH2) | PM 4360 |
| IF (THZ . 7.3.$)$ EKC FEKCS(TH2) | PM 4370 |
| IF (TH2 GT: 10, EKC PEKCHETH2) | PM 4380 |
| EKS FEKS (TH2) | PM 4390 |
| IF (THE LT, 3, EKS E FEKSS(TH2) | PM 4400 |
| IF (TH2 GT, 10, EKS EEKSH(TH2) | PM 4410 |
| EK20R FEK2OLPH2) | PM 4420 |
| IF (TH2 .6T. 3, EK2DR EFEKZOS (THZ) | PM 4430 |
| TF (TH2 GE Q E EKZDR EEK2OU(TH2) | PM 4440 |
| IF TTH2 GF. 10.) EKZDR (FEK2DH(TH2) | PM 4450 |
| EKCSAVE (EKC+EK8)/3. | PM 4460 |
| EK2DC EK2DR,EKC/EKS | PM 4470 |
| $E K 2 D C S$ (EK2DR+EK2DC7/2. | PM 4480 |
| IF (ISHAPI .NE, 1 OR, ISHAPZ .NE, 1) GO TO 216 | PM 4400 |
| C.E | PM 4500 |




| C.0. | $\begin{aligned} & \text { PM } 5510 \\ & Q_{M} \quad S 520 \end{aligned}$ |
| :---: | :---: |
| C.: TEST SECTIONS | PM 5530 |
| c | PM 5540 |
| C. ${ }^{\text {co }}$ | PM 5550 |
| Cen CLOSED, CONETANTEAREA IEST SECTION | Pr. 5560 |
| C.O | PM 5570 |
| 1010 EK SLMPAILELIOHI | PM 5580 |
| EKO EEKO (EK, AO, A1, EMO, AMACH\&, EMOSO, 6 ) | PM 5590 |
| SEKO(N) EKKO | PM 5600 |
| GUMEKO SUMEKO\#EKO | PM 5610 |
| C-CALL OUTPUY(1,1) | PM 5620 |
| LINECT LINECT+3 | PM 5630 |
| IF (ISEC, E0, 1) 60 10 200 | PM 5640 |
| C): | PM 5650 |
| Con MOnEL IN THE IESLEECLION | PM. 5660 |
|  | PM 5670 |
| $820 \mathrm{gOA}=\mathrm{DATA}(8)$ | PM 5680 |
| CD DATA (13) | PM 5690 |
| $E P_{s}$ - $1 .+D A T A(16) / 100$. | PM 5900 |
| EK = CD*SOA*EPS | PM 5710 |
|  | PM 5120 |
| SEKO (N) E SEKO (N) QEKO | PM 5730 |
| SUMEKO EUMEKOtEKO | PM 5740 |
| CALL OUTPUT(2,3) | PM 5750 |
| LINECT LINECT+3 | PM 5760 |
| 60 TO 200 | PM 5770 |
| Ces | PM 3780 |
| C.O CLOgED, OLFFUgING TEST gECTION | PM 5790 |
|  | PM 5800 |
|  | PM 5810 |
| IF EK LT. CSLMDALHELTOHIJLEK SMOAIEELOHI | Pm 5820 |
| EKO FEKO EK, AO, $12, E M O, A M A C H 1, E M O S U, ~ O) ~$ | PM 5830 |
| 8EKOSND EKO | PM 5840 |
| SUMEKO EUMEKO EKO | PM 5850 |
| CALL OUTPUT(3,2) | PM 5860 |
| LINECT LINECT*3 | PM 5870 |
| IF (IsEC.E日. 3) 60 TO 200 | PM 5880 |
| 60101020 | PM 5890 |
| Cen | PM 5900 |
| C.O OPEN=THROAT PE8T SECTION | PM 5910 |
|  | PM 5920 |
| 1050 EK .0y45*EL/OH10.0053*(EL/DH1)**2 | Pr 5930 |
| $\text { EKO EEKQ(EK,AO,AI,EMO,AMACMI,EMOSQ, } 6$ | PM 5940 |
| SEKOR \% EKO | PM 5950 |
| SUIAEKO SUMEKCEKO | PM 5900 |
| TH2 = FTH2 (A2,A1, EL) | PM 5970 |
| CALL QUTPUT(3,4) | PM 5980 |
| LINECT E LINECT+3 | PM 5990 |
| IF (ISEC.E日, 5) 60.10200 | PM 6000 |



| $1330 \text { NN NH: ABSCOATA(11) }$ | PM 6510 PM 6520 |
| :---: | :---: |
| EKTE90 OATA (12) | PM PM P OS |
|  | PM 6540 |
| EKTE FKTE (PHI)*EKTE9011.80 | PM 0550 |
| EK CEKTE\&BLMDAL*EL/OHL | pm 6560 |
| EKO FEKO (EK, AO, AL, EMO, AMACHI, EMOSG, 0 ) | PM 6570 |
| ITYPE 1 | PM 6580 |
| 00102000 | PM 6590 |
| C. | PM 6600 |
| C.: DIFFUSING CORNER WITH TURNING VANEg AND WALLS | PM 6610 |
|  | PM 6620 |
| NN 10 <br> CHORD : DATACO | PM 6630 PM 6640 |
| PH m AS (DATA (li)) | PM 6650 |
| EKTV90 E Datallej | PM 6660 |
| RNREF OATA (14)*10***6 | PM 6670 |
| IF (EKTV90. 1 T. 1.EOU) EKTVOO -. 15 | PM 6680 |
|  | PM 6690 |
| IF (PHI LEE - 30.1 EKTV E FKTVI(PHI) *EKTV901. 15 | PM 6700 |
|  | Pmi 6710 |
|  | PM 6720 |
| RNV: RNOC*CHORU/A1*(1.*(G.1.)/2,*AMACH1**2)***96 | PM 6730 |
| $E K V=3$ | pm 6740 |
|  | PM 0750 |
| Ex - EKV* ( $A R=1.1 / A R) * * 2$. | PM 6760 |
| EK2 EKKTV ${ }^{\text {c }}$, ${ }^{3}$ | PM 6770 |
| CALL FHICTNSCHORD, A, AMACHI, SLAMDAI | Pm 6780 |
| EK EK EK ${ }^{\text {EK2 }}$ | PM 6790 |
| EKO = FEKOREK, AO, A1, EMO, AMACHL, EMQSO,G) | PM 6800 |
| 1TYPE 3 | PM 6810 |
| 60903000 | PM 6820 |
| C.E... | PM 0830 |
|  | PM 0840 |
| c. | PM 6850 |
| C.O OFFUSER | PM 6860 PM 6870 |
|  | PM 6880 |
| 400 NN $=11$ | PM. 6890 |
|  | PM 6900 |
|  | PM 0980 |
|  | PM 0921 |
| ITYPE ${ }^{\text {a }} 3$ | PM 6930 |
| GU TO 2000 | PM 6440 |
| C.: EXIT KINETIC ENERGY FRO | PN 6950 |
|  | PM 6900 PM 6970 |
| $1450 \mathrm{NN}=12$ | PM 6980 |
|  | PM 6990 |
| EKO EEKOEE,AO, A1, EMO, AMACH1, EMQSE,G) ... . | PM 1000 |



| $\begin{aligned} & \text { EKCNTR : }{ }^{32 \star E L C / O H L \star S L A M O A} \\ & \text { IHZ FYH2(AZ,AL,ELD) } \end{aligned}$ | $\begin{aligned} & \text { PM } 7510 \\ & \text { PM } 7520 \end{aligned}$ |
| :---: | :---: |
| $\text { Exy }=3$ | PM 7530 |
|  | PM 7540 |
| EKD EKV*((AR-1, )/AR)**2 EK EKCNTR\&EKD | PM 7550 PM 7560 |
| EKO FEKO (EK, AO, A1, EMO, AMACHI, EMOSO, G) | PM 7570 |
| ITYPE E 3 | PM 7580 |
| 60102000 | PM 7590 |
| C-0-_- | PM 7600 |
| C:\% PERFORATED PLATE WITM SHARPDEDGED DRIFICES | PM 7610 |
| c | Pm 7620 |
| $1530 \mathrm{NN}=16$ | on 7630 |
| PRSTY DATH(10) | PM 7640 |
|  | om 7650 |
| EKO FEKOCEKAARALEEMR.AMACMLEEMOSQ.G1 | PM 7660 |
| ITYpE 2 | PM 7670 |
| 69102000 | PM 7680 |
| C.: | PM 7690 |
| Ce. WOVEN MESM SCREEN | PM 7700 |
|  | pr 7710 |
| 1540 NN E 17 | PM 7720 |
| UMESH E DATA(9) | PM 7730 |
| ORSTY = DATA1101 | Pm 7740 |
| EKMESH DATA(12) | PM 7750 |
| IF (EKMESH LT 1, E00] EKMESH $=1.3$ | PM 7760 |
|  | PM 7770 |
|  | PM 7780 |
| ENU EMU/AHOS | PM 7790 |
| KN V1. DNESH/ENU | PM 7800 |
| EK EKMESH* (1.0PRSTY/100, + (100, /PRSTY-1.) **2 | PM 7810 |
| IF 1 RN LT, $400.4 N D$ | PM 7820 |
|  | PM 7630 |
|  | PM 7840 |
| EKOEFEKO (EK, AO, A1, EMO, AMACHI, EMOSG,G) | PM 7850 |
| $I T Y E=2$ | PM 7860 |
| GO 10 2000 | PM 7870 |
|  | PM 7880 |
| C.. INTERNAL STRUCTURE (DRAG ITEM(S)) AT UPSTREAM END OFA | Pr 7890 |
| CAM-_- SEETION | PM 7900 |
| C: | pr 7910 |
| -1560 NN $=18$ | Pm 1920 |
| ENITEM E OATA(2) | PM 7930 |
| SOA E OAIA (B) | PM 7940 |
| $C D=O A T A(13)$ | PM 7950 |
| EPS | PM 7960 |
| IF (ABS(ENITEM) , LT, 1,E®6) ENITEM a d, | PM 7970 |
| EK ED.SOA,EPSHENITEM | PM 7980 |
| EKO FEKDIEK, AO, AL, EMO, AMACHI, EMOSO, G) | PM 9990 |
|  | PM 8000 |








```
    AVGPWR PWROP/ENFAN PM11010
    NRITE(O,9303) SUMEKOEER,OWRIP,PWROP,AVGOWR,EIAFAN,ENFAN
    LINECT E GINECT+5
```



```
    PM11020
    Pm&1030
```



```
C....PUWER VALUE - DETERMINES APPROXIMATE MAXIMUM TEAT SECYION VELOCPY
C.,#FGR THE SPECIFIED POWER LEVEL
C...
    CALL FRICTNIOMORADEMD,SLMDACI
    vOC v vo
```



```
    EMF E EMF WO/VOC
    Em0 GMOLYO/VOC PM11140
    RHOSF RHOT/(1.+(G-1.)/12.#EMF**2)**(1./(G-1.)) PM11150
```



```
    RNINE RHOSO*VO*AO/EMUT FM11170
```




```
    1 RMOSFIETAFAN
```




```
    IF (ABS((PWRMCH-PNROP)/PWRMCH),GT, 1,E~6) 60 TO 2009 PM11230
```



```
    IF (IJ.FW, 2) VOK = VO*.5924B PM11250
```



```
    IF (LINEET LY, (LINEMX-10)) GO TO 2010 PM&&270
```



```
    WRITEPG,9001, ITITLE,IPAGE PMI1290
```



```
    2011 IF IPRINI,EQ. Q1 GOTD 2014 _
```



```
CB:RGIRGILSUMMARY GHARACIERISILCS PAGE OLIPUI__ PMII350
```



```
    D02n13 I & I,N 0M1/370
    IF ITINECT ITTEINEMXI_GO TO 2012 _ PM11380
    IPAGE IPAGE+! PM11390
```



```
    WRITE(0,9401)
    IF (IU.ER. 1) NRITE(6,9402) PM11420
    IF IIU EG. 2) WRITE(0,9403) PM11430
    B1ECT:15 PM11440
    20IZ WRITE(6,9400,I,SSUMEL(I),SMACH(I),SSUMKO(I),DELP(I) PM&1450
```



```
    2013 CONTINUE
    2013 conTINUE
    PM11470
```



```
C.O.......' PM18490
```


C.". CALG PLOTITEPAELDESSUMEL,SSUMKU,IUAIPLOTAITITLE, IRETRN, PLOTON) C.A.ABMUFATER TAUULATIUN OF INPUT DATA EAYOS FUR CURREAT CASE

PM11510
PM11520
Pッ11530
PMi1540
PM11550


PMI 1560
PM11570

C... PM11590

WHITE(0,9007) ITIILE
IF (TGETRN,GT. G E=6) GOTO 100 OM11010
IF THETRN.LT. .5, GQ 10.102
STOF 1060
Pm1 1630
$c$
C***********************************************************************
 PM11640
C. INOUT REAU FARMATS

Cfiñ furmat (a1,19A4,a3)

7 JOL FUKMAT (212,211.2f2.0.14F5.2)
c, equegengectuses.
C. oduput formats
C.:

PM11650
PM11650
PM11060
PM11670
PM11680
PM11690
PM11700
PM11710

PM11750
C.EAEKRFORMANCE INFORMATIUN LABELLING ANO OUTPUT FORMATS

PM11720
0M11730
P411740
PM11760
Cgigo plormat flHf/f2OX,A1,19A4,A3,13X,4MPAGE,I3//
1 20x:944,24m $\times 1 N D=T U^{9 N N E L ~ P E R F O M A N C E /, ~}$
PMII 1770
PM11760
PM11790
PM11800
PM11610
PM11820
prilis30
Pwil1840
Pmil1850
PM11800
PM11870
PM11880
PM11890
PM11900
P411910
Pm11920
Pul1930








ATERMINATION CODE, THE INDUT POWER VALUE IS ILLEGAL (LESS THAN OR/ PMI3OIO B $120 H$ EQUAL TO ZEROS. IHEREFORE, NO VELOCIIYOOPTIMIZING IS PO PMIBOZQ CSSIBLE, RECHECK INPUT VALUE ON MASTER DATA CARD, PM13030 Q509 FORMAT L/L15H t ERROR E SOAE INCOREEGT COMBINAYION OF INPUIS OR PMI 3040 AUNANTICIPATED SITUATION HAS CAUSED AN INVALID (NON-POSITIVE) 1 PM\& 3050 Q 39H TOTAL LOSS LEVEL RECHECX SECIION, IJ. 12H INPUT DATA.LI PMI306O EN

PM13070




| $\begin{aligned} & \text { DOR2001 : } 5,20 \\ & \text { ENDATA(N,I) DATA(IG4) } \end{aligned}$ | $\begin{gathered} \text { OK } 1510 \\ \text { OK } \\ 1520 \end{gathered}$ |
| :---: | :---: |
| 2001 CONTINUE | OK 1530 |
| Comer | OK 1540 |
| C.G.GENERAL INPUT REQUIREMENT DEFINITIONS | ok 1550 |
| C.a | OK 1560 |
| DO 2002 1 1,20 | DK 1570 |
| NCHECK(NaI) 0 | OK 1580 |
| 2002 CONTINUE | OK 1590 |
| NCHECK(N, 1) ? | Ok 1600 |
| NCHECK(N, 2) 1 | DK 1010 |
| NCMECK $(N, 3)=1$ | OK 1620 |
| NCHECK (N, 4) E1 NCHECK | OK 1630 |
| IF (1SHAP1, NE, 1) NCHECK (N, 7) \& | OK 1640 |
| NCHECK ${ }^{\text {N, }}$, 8) 1 | DK 1650 |
| NCHECK(Ne 2$)$ | OK 1660 |
| IF (ISHAP2 ${ }^{\text {NE, }}$ (1) NCHECK(N,10) = 1 | DK 1670 |
| NCHECK(N,11) : | $\begin{array}{ll}\text { DK } & 1680 \\ \text { OK } & 1090\end{array}$ |
| C.A.ASECYION.TYPE BRANCHING | OK 1700 |
|  | DK 1710 |
| IF CISEC IES, 1) 60 T0 3000 | OK 1720 |
| IF (18EC.E日, 2) 60102020 | ok 1730 |
| If (ISEC EQ 3) 60102030 | DK 1740 |
| IF (ISEC , En, 4) 60902040 | ok 1750 |
| If PI8EC, EQ, 5, 60103000 | OK 1760 |
| IF (ISEC Ed, 0) 60102000 | OK 1770 |
| IF CISER, EEQ - 10 ) 60 T0 3000 | - DK 1780 |
| If (ISEC.Eq. 20) 60 T0 3000 | OK 1790 |
| IF-18EC.EQ 30, 60 102300 | ak 1800 |
| IF (18EC EEQ, 32) G0 102300 | DK 1810 |
| IF 1SEC,EQ, 33, 60 T0 2330 | DK 1820 |
| IF (1SEC .EQ, 34) 60102340 | OK 1830 |
| IF CI8EC EEQ 40, 60 T0 2400 | DK 1840 |
| IF (IsEC .E®, 45) 60 T0 2450 | OK 1850 |
| IF (ISEC EQ, 40) 60 T0 2460 | OK 1800 |
| IF (ISEC .ED. 51) 60 TO 2510 | DK 1870 |
| IF fSEC, EQ 52, 60 T0 2520 | DK 1880 |
| IF (1SEC .ED, 53, 60102530 | OK 1890 |
| IF (15EC.E日, 54) 60 10 2540 | OK 1900. |
| IF (IGEC , EQ, 50) 60 O 2500 | Ok 1910 |
| 1F (ISEC.EQ, 57, 60 to 2590 | OK 1920 |
| NCHECK(N, 5) 1 | DK 1930 |
| IF (ISEC, EQ 61, 60 103000 | OK 1940 |
| IF (I8EC.EQ. 62) 60103000 | OK 1950 |
| IF CISEC, EQ: 10 LG0 10 2700 | EK 1.960 |
| IF (IsEC , EQ, 71) 60 TO 2700 | DK 1970 |
| IF (SSEC,E0, 72) 60.102700 | DK 1980 |
| IF (ISEC .EG, 73) 60 10 2730 | OK 1990 |
| IF (ISEC.EQ. 14) 60 102740 | OK 2000 |







| $\begin{gathered} \text { 8EAROR -2. } \\ 3002^{\text {BCONTINUE }} \end{gathered}$ | $\begin{aligned} & \hline \text { OK } 4510 \\ & \text { OK } 4520 \\ & \hline \end{aligned}$ |
| :---: | :---: |
| c..... INVALID SECTION SHAPE CHECK AND MESSAGE | DK 4530 |
| Con InYaLid secilon shape check and MEssage | Or 4540 |
| ${ }^{\text {cjojos }}$ | $\begin{aligned} & \text { OK } 4550 \\ & \text { ok } 45 \mathrm{hog} \end{aligned}$ |
|  | OK 4590 |
| IPAGE LPAGE+1 | OK 4580. |
| WRITE(6,0111) ITITLE,IPAGE | OK 4390 |
| WRITE 6,8007 ) | -0x 4 eno |
| If (IU ,EQ, 1) WRITE(0,8008) | OK 4010 |
| IF IU .ER, 2) WRIIE(0,8009) | OK 4620 |
| WINECT 3 a | OK 4030 |
| 3004 WR1TE 6.8003$) \mathrm{N}, 18 \mathrm{HAPI}$ | OK 4640 |
| LINECT - LINECT+3 | OK 4650 |
| 8ERROR ${ }^{\text {a }}$-2, | OK 4660 |
|  | DK 4670 |
| IF SLINECT LT, (LINEMX-2) 60 90 3006 | OK 4880 |
| IPAGE IPAGE | OK 4690 |
| WRITE(6,8111) ITIFLE, IPAGE | OK 4700 |
| WRITE (6,8007) | OK 4710 |
| IF (IU CEQ. () WRITE $6,800 \mathrm{~B})$ | OK 4220 |
| IF (IU -EG. 2) WRITE ( 0,8009 ) | OK 4730 |
| LINECT ${ }^{\text {a }}$ - | DK 4740 |
| 3000 WRITE (6, 0003 ) NOISHAP2 | OK 4750 |
| LINECT: LINECT+3 | DK 4760 |
| SERROR -2. | OK 4770 |
| 3007 IF (8ERROR GT, - - 10.60 IO 3009 | 0K 4780 |
| IF (LINEC .LT, (LINEMX=3)) 60103000 | DK 4790 |
| IPAGE - IPAGE ${ }^{\text {P }}$ I | DK 4800 |
| WRITE (b,AIII) IPIPLE,IPAGE | OK 4810 |
| WRIIE 6,0007 ) | DK 4820 |
| ${ }_{\text {IF }}$ (IU EEO. !) WRITE $(6,8008)$ | OK 4830 |
| IF (IU EEC. 2) WRIE (6,B00.9) | DK 4840 |
| LINEC 0 | OK 4850 |
| 300日 WRITE (0.8005) N | OK 4800 |
| LINECT - LINECT 3 | nk 4890 |
| 3009 IF (TLIST. GTR SERROR) TLIET E SEROR | DK 4880 |
| RETURN | DK 4890 |
| C | DK 4900 |
|  | OK 4910 |
|  | OK 4920 OK 4930 |
|  | DK 4940 |
| c....isidelina | OK 4950 |
| C.AR.RALELGING OF MABTER ANO TERMINATION DAIA Page | OK 4960 |
| 4000 IPAGE - IPAGE+1 | DK 4980 |
| WRITE(6,8100) ITITLE,IPAGE | DK 4990 |
| * *RIE (0,0101) | Or 5000 |



| 0040141 I 6,131F (MCHECK(I), NE, O) 00 10 4008 |  |  | $\begin{array}{r} 5510 \\ 5520 \\ \hline \end{array}$ |
| :---: | :---: | :---: | :---: |
|  | EMWRIT (IOV) RBLNK2 | DK | 5530 |
|  | EMWRII(IOY +1) RELNK4 | OK | 5540 |
| IF (AES (EMDAPA(I)EMWRIT LOV) LT, 1,EDG) OO YO 4012 |  | OK | 5550 |
|  |  | OK | 5560 |
| EMWRIP(IOV+1) RMSG2 |  | DK | 5570 |
| 00104012 |  |  | 5580 |
|  |  |  | 5590 |
|  |  | OK | 3600 |
| c. IOV IOV+1 |  |  | 5610 |
|  |  | OK | 5620 |
| C.e- OATAMMAGNITUDE =CONTROLGED FORMATTINO |  | OK | 5630 |
| c. |  | OK | 5640 |
|  | MFURMY(IOF) I IFFLDO |  | 5650 |
| IF (EMDATACI) LLT 1000. MFORMTCIOF) IFFLDI OK 5460 |  |  |  |
| IF (EMDATA(I) 1FT. 100.) MFORMT(IOF) EIFFLDZ <br> IF (EMOATA(I) LT 10.) MFORMT IOF) E IFFLDS |  | OK | 5670 |
|  |  |  | 5680 |
| IF (EMOATA(I) , WT. $1:$ ) MFORMP(IOF) IFPLDU |  | OK | 5690 |
| MFORMI (IOF + 1) [ ICOMMA |  | OK | 5700 |
|  |  | OK | 5710 |
|  |  |  |  |
| EMWRIT(IOV) RMSG3 |  | OK | 5730 |
| EMWRIT (IOV 11$)$ RMSO4 |  | OK | 5740 |
|  | 60 10 4012 | OK | 5750 |
|  |  |  |  |
|  |  |  |  |
| EMWRII(IOY +1) RMSC6 |  | OK | 5780 |
|  |  | DK | 5790 |
|  |  | DK | 5800 |
| EMWRIY(IOV+1) RMSGO |  | OK | 5810 |
| 4012 10V - IOV.2 |  | DK | 5820 |
| MFORMP (10F) IAFLDE |  | OK | 5830 |
|  |  | OK. | . 5840 |
| 4013 IOF = 10F +2 |  | OK | \$850 |
| 4014 | CONIINUE | OK | 3860 |
| IOV $10 \mathrm{~V}-1$ |  | OK | 5870 |
|  |  |  | 5880 |
| C... MEPENATION OF TERMINATION CONPROL CODES |  | OK | 5890 5900 |
|  |  | OK | 5910 |
|  |  | DK. | 5920 |
| C. SUMMARY INFORMATION PRINT |  | OK | 5930 |
|  |  | OK | 5940 |
| ETWRIT(1) TMSGZ <br> IE CIPRINT NE OLETWRIIL IL TMSGL |  |  | 5950 5960 |
|  |  |  |  |
|  |  |  |  |
|  |  |  |  |
|  |  |  |  |

```
    ETHRIT( 2) = RELNKA OK 0010
    ETWRIT( 32 ETMSG3 _
    ETWRIT( 4) = RBLNK4 OK OO3O
```



```
    4015
    IF (IPLOT NE, I ANO, IPLOT ,NE, 3) 60 TO 4016 OK 6050
```



```
    ETWOIT( 3) =TMSG5 OK 0070
    ETWRITP 4) ETMSGOA
    IF IIPLOT,GT, 2) ETWRIT(4) E TMSGGO OK 6090
```



```
4010
    ETWRIT( 2) TMSG7 OK O1&0
```



```
    ETMRIT(4) ETMSGQ OK 6\30
```



```
    U0 40181 1 10,12 OK 6150
```



```
    GO TO 4020 OK 6170
```



```
    ETWGIT(1L):TMSG8 OK O190
```



```
C..... OK G210
```



```
    C002O ETWFIT( 5) E TMSEL
    ETWRIT(5) EMSG1
    ETWRIT( 0) TMgGL2
        OK 6240
    DN 0250
    ETNRIT( 7) TMSGI3 DK 6270
```



```
    4021 ETWAST(6) # TN$G10 OK O290
```



```
    4O22 ETWRIT( B) FHSG2 DK 6310
C.e.es... OK 6320
C.: POWER_MATCHING AND VELOCITY.OPTIMIZATION REQUEST
DK }633
    IF (PWRI .GT, 1,E=0) ETWRIT, 8) E TMSGI DK. 6340
    ETMQIIGQ) IMSG2 DK 6360
C.0.", OK 6370
CEA NEXT-CASE,RETURN OR TERMINATION REQUEST OK 6380
C.O IF (TRETRN GT. 1-E=6) ETHRITE Q) TMSG, DK 6390
```



```
    WRITE(0,B105) N
    WRIFF(0,8100) ETWRIT
OK 6410
```



```
C....##̈̈g
C.ORHEADINGS FOR LISTING OF SECTION DATA INPUTS OK 6440
C... OK O450
```



```
    WRITE(6,8100) ITITLE,IRAGE OK O470
    WEITS(6.8107)
    IF (IU EG. 1) WRITE (0,8108) OK 0490
```




|  | 10VI - IOV-1 | DK | 7010 |
| :---: | :---: | :---: | :---: |
|  | IOVA | OK | 7020 |
| C..... |  | DK | 7030 |
| C.e- | FLOATINGPPOINI INFORMATION | OK | 7040 |
| C. |  | OK | 7050 |
|  | 00 4038 3 : 5,20 | OK | 7000 |
|  | IF (NCHECK(I, J) NE, 0) 60 T0 4031 | OK | 7070 |
| E | ENWRITIIOV) HBLNK2 | DK | 7080 |
|  |  | OK | 7090 |
|  |  | OK | 7100 |
|  | ENHRIT (IOV) RMSGI | OK | 7110 |
|  | ENWRIT (IOV迷) RMSG2 | OK | 7120 |
|  | \%O T0 4036 | OK | 9130 |
| 40311 |  | DK | 7140 |
|  | go 104033 | OK | 7150 |
|  | ENWRIT(IOV) ENDATA(Lej) |  | 7100 |
|  | 10V $10 \mathrm{~V}+1$ | OK | 7170 |
|  | IF J , NE, S AND, J, NE, ف, GO PO 4032 | DK | 7180 |
|  | NFORMY (IOF) E IFLDO | DK | 7190 |
|  | NFORMT (IOF +1$)$ ISPACC | DK | 7200 |
|  | GOTO 4n3 | DK | 7210 |
| Corese |  |  | 1220 |
| C. ${ }^{\text {c }}$ | DATA.MAGNITUOE.CONTROLLED FORMATYING | DK | 7230 |
|  |  | DK | 1240 |
| 4032 N | NFORMT (IOF) = IFFLDO | DK | 7250 |
|  | IF (ENDATA (LeJ) LT. 1000.) NFORMT (TOF) - FFFLOI | DK | 7200 |
|  | IF (ENDATA(L,J) LT. 100. ) NFORMT (IOF) IFFLD | OK | 7270 |
|  | IF (ENDATA (INS) LIE HRS NFORMY(IOF) - IFFLOS | DK | 7280 |
|  | IF (ENDATA(I, J) LT, 1, NFORMT (IOF) IFFLDA | OK | 7290 |
|  |  |  | 7300 |
|  | 60 T0 4039 | OK | 7310 |
| 40331 |  | OK | 7320 |
|  | 60904034 | OK | 7330 |
|  | ENWRIT(IOY) ERMSG3 |  | . 7340 |
|  | EVWRIY(IOV+1) RM864 | OK | 7350 |
|  | 60104036 |  | 7360 |
| 40341 |  | DK | 7370 |
|  | O0 T04035 . An | OK | 7380 |
|  | ENWRIT (IOV) = 9 M 865 | OK | 7390 |
|  | ENWRII (10Y+1) : RMSG6 | OK | 7800 |
|  | G0 104036 | OK | 7410 |
| 4035 | ENWRIT (IOV) E RMSGT | OK | 7420 |
|  | ENWRIT (IOV+1) RMSG8 | DK | 7430 |
| 40361 | $10 y-10 y+2$ | OK | 7440 |
|  | NFURMT (IOF) IAFLDE | OK | 7450 |
|  | NFORMT | OK | 7460 |
| 403710 | IOF - IOF +2 | DK | 7490 |
| 4038 | CONTINUE | 21 | 7480 |
|  | IOV $10 \mathrm{~V}=1$ | OK | 7490 |
|  | NRITE (Q,NFORMT) (NWRITE(J), | OK | 7500 |





 END OK 8540


| SUBROUTINE FRICTN(OH, A, AMACH, SLAMDA) | PN 10 |
| :---: | :---: |
|  | PN 20 |
| C*********************************************************************** | FN 30 |
| C IHIS ROUTINE, SUBROUTINE OF THE MAIN PRORRAM PERFORM, COMPUIES IHE | PN $\quad 40$ |
| C LUCAL REYNOLDS NUMEERS ANO SMOOTHEPIPE FRIETION COEFFICIENTS. | PN 50 |
|  | FN 60 |
| C******れ************************************************************** COMMON/RLOCKC/ ASTAR,AT,G | $\begin{array}{ll} P N & 90 \\ P N & 00 \end{array}$ |
| COMMON/RLOCKD ${ }^{\text {RNOC }}$ | FN ${ }^{\text {P }}$ |
| Come | HN 100 |
| C. REYNOLOS NUMBER BASED ON THE CHARACTERISTIC DIMENSION DH CUSUALLY BUT | FN 110 |
| CENOT ALWAYS IME HYORAULIC OIAMETER OF IHE LOCAL DUCTI | N $\mathrm{N}-120$ |
| C... | FN 130 |
|  | PN 140 |
| IF (RN, GE, 4,E3) GO TO 1 | PN 150 |
|  | EN 160 |
| C.ORICTION COEFFICIENT | FN 170 |
|  | FN 180 |
|  | F $N 190$ |
| CAP PFOR REYNOLDS NUMAERS LESS THAN 4000 | FN 200 |
| C. . ${ }^{\text {a }}$ | FN 210 |
|  | EN 220 |
| C..."EAR'REYNOLOS NUMRE | $\begin{array}{ll}\text { FN } & 230 \\ \text { FN } & 240\end{array}$ |
| C.: | FN 250 |
|  | EN-260 |
| 60904 | PN 270 |
| Ceneshenere | FN 280 |
| C..., NEMTUNIS METHOD ITERATION FOR FRICTION COEFPICIENT AT REYNOLDS | FN 290 |
| C.GOSUMBERS GREATER THAN OR EQUAL TD 4000 | PN 300 |
| C... | - N 310 |
| 1 SLAMT 005 | +N 320 |
|  | FN 330 |
|  | HN 340 |
| IF ( $\triangle$ SS(SLAMN=SLAMT)/SLAMN LT, 1,E由4) 60 ¢0 3 | FN 350 |
| SLAMT SLAMN | PN 360 |
| GO TO 2 | FN 370 |
| 3 SLAMDA SLAMN | FN 180 |
| 4 RETURN | FN 390 |
| ENC | FN 400 |





```
        DATA NSECT( 96)ONSECT( 97),NSECT( 9B)ONSECT( 99),N8ECT(100)/ OT 530
```



```
        DATA NSECT(10!),NSECT(IOZ),NSECT(103),NSECT(104),NSECT(105)/, OT 550
    1. 4MMULI, UH DUC, 4HT CO, UHNTRA, 2HCT/ - OF 560
        UATA NSECT(100),NSECT(107),NSECT(108),NSECT(109),NSECT(110)/ OT S70
```




```
        OATA NSECT(116),NSECT(117),NSECT(118),NSECT(119),NSECT(12V?/ OT b10
        1 4MMUT, 4HO 2, 4HOWA1, 4HL CR, 2HNR, OF 62O
```




```
    1, LHM D &H ERN, 4HR, N, 4HO VA, 2HNE/ 
```





```
    1 GHMULT, 4H DUC, 4HY DI, GHPFUS, 2HER/ 
        CATA NSECT(141),NSECT(142),NSECT(143),NSECT(144),NBECT(14S)/ 0% 710
    L
    i
```



```
        I UHSUD, 4HEXP, 4HM O 4HE SN, 2HGL/ 
    1. HHSUN, 4MEXP, 4HMD,4HEM, 2HD, OT 760
```




```
        DATA NSECT(161),NSECT(162),NSECT(163),NSECT(164),NSECT(105)/, OT 790
```



```
        DATA ASECT(166),ASECT(167),NSECT(168),NSECT(169),NSECT(170)/ OT B&0
    & GHFAN, GHNIFS, 4HR&CN, 4MTR G, 2HDY/
        DATA NSECT(17!),NSECT(172J,NSECT(173),NSECT(174),NSECT(175)/
    1. GWMULT, GM INI, GHENL, UHSIRE, 2HIR/
        UATA NSECT(176),NASET(177),NSECT(178),NSECT(179),NSECT(180)/
        1 4MMULT. UHIPL, 4HFIXE, 4MD LO. 2HSS/
C...."OEOIIOMN END-SHAPE NAME DEFINITIONS
C....OBOCIIONN ENDGSHAPE NAME DEFINITIONS
C...
```



```
        N1 = NN+5=4
        N = U1+4
        IF (ISHAPI ,NE, ( OR, ISHAPZ NE, 1) GOTO &
CHORTTE*SHTATEMENTSGODR SECTIONS WHICH HAVE CIRCULAR CROSSESECTIUNS
C.->AI_BOIM ENDS
C...
        IF (ITYPE,EO.-1.)
        I HRITE(O,OIII) N,(NSECT(I),IONI,NS),NSMAPE(ISHAPI),DI,M1,M1OAU,
    2. Y1,AMACH1,EK,EK,EKO,NSHAPESISHAP2, O2,A2,AZOAO,Y2,AMACH2
Of 820
        AT 840
        or 850
        OT 800
        ता 870
            OT BBO
OT 890
*.0
    0% 900
```



```
NECT(74),NSECT(175)/ of 83
```



| $\begin{aligned} & 3 \text { EK, EKO } \\ & 60 \text { To } 4 \end{aligned}$ | $\begin{gathered} \text { of } 1510 \\ \text { of } 1522 \end{gathered}$ |
| :---: | :---: |
|  | of 1530 |
|  | OT 1540 |
| C..AT THE UPSTREAM END AND CIRCULAR CROSS.BECTION AT THE DOWNSTREAM | 0 OT 1550 |
|  | of 1560 |
|  | of 1570 |
| 1F (ITYOE EQA. 1) | 0) 1580 |
|  | 011590 |
|  | 011600 |
| IF (ITYPE E0, 2) | OT 1610 |
|  | 071620 |
| 2 A1OAO, V1, AMACH1, EK, EKO,NSHAPE (ISHAP2), D2, A2, A20AO, V2,AMACH2 | -1 1630 |
| IF (ITYPE,EO, 3) | or 1640 |
| 1 WrITE(6,9221) N, (NSECT(I),IEN, N5), N8HAPE (18HAP1), M1, W1, A1, |  |
| 2.ALCAO, AR, TH2, Y1, AMACH1, EL, EK, LKO, NSHAPE (ISHAP2), D2, AR,AZO10, V2, | 011660 |
| 3 AMACH | DT 1670 |
| IF (JYPE,EQ 4) | of 1680 |
|  | Of 1690 |
|  | 01.1200 |
| 3 Ek, EkO | of 1710 |
| REIURN | of 1120 |
|  | Of 1730 |
| Ceies SECiIION PERFORMANCE CALCULATION OUTPUT WRITE PORMATS | 011740 |
|  | Of 1750 |
|  | of 17760 |
|  | $\begin{gathered} \text { of } 1770 \\ \text { of } 1700 \end{gathered}$ |
|  | of 1790 |
|  | 011800 |
| A $59,2,259,5 / 23 x, A 4,8 x, 59,2, F 11,2, F 7,2,10 \times, 78,1, F 7,3)$ | Of 1810 |
|  | ot 1820 |
| A F9, 2 /23x, $44,8 \times, F 9,2, F 11,2, F 7,2,16 x, F 8,1,57,3,9 x, 250,5)$ | 011830 |
|  | 011640 |
| 4 $59,2,279,5 / 23 x, 14, F 8,2, F 9,2, F 11,2, F 7,2,16 x, F 8,1, F 7,3$ ) | DT 1850 |
|  | OT- 1860 |
|  | OT 1870 |
|  | D1 1880 |
|  | DT 8180 |
|  | OT 1900 |
| A $79,2123 \times, 44, F 8,2,79,2,511,2,57,2,16 \times 1.58,1,57,3,9 \times, 2 F 9,51$ | OT 1910 |
|  | OT 1920 |
| A 2F9, 5/23x, A4, F8, $2, F 9,2, F 11,2,57,2,16 x, F 6,1,54,31$ | or 1930 |
|  | of 1940 |
| A 2F9, $5123 \times, 44,78,2, F 9,2, F 11,2,57,2,16 x, 78,1,57,31$ | OT 1950 |
|  | -1 1960 |
| A F9, 2, 2F9,5/23x, A4, F8, 2, F9, 2, F11,2,F7, $2,10 \times, F 8,1, F 7,3$ ) | of 1970 |
| 9192 FORMAT $1 / 13,1 \times, 4 A 4,42,1 \times, A 4,8 x, F 9,2, F 11,2,2 F 7,2, F 9,2, F 8,1, F 7,3$, | Of 1980 |
| 4 F9, 2/23x, A4, FB, 2, F9, 2, F11, 2, F7, 2, 10x, FE, 1, F7, 3, 9x, 2F9,5, |  |
|  |  |

```
    AF9,2,2F9,5/23x,44,8x, 10,2,F11,2,F7,2,16x,F8,1,F7,3) OT 2010
```



```
    A 2F9,5/23x,A4, 8x,F9,2,F11,2,F7,2,16x,F8,1,F7,3, 
```



```
    AF9,2,2F9,5/23x,44,8x,F9,2,F11,2,F7,2,16x,FB,1,F7,3)
```



```
    A F9,2/23x,44,8x,F9,2,F1,i,2,F7,2,16x,F8,8,7%,3,9x,2F9,5,
    OT 2070
    END
    01 2080
```

| ```SUBROUTINE PLOTITCN,DELP,SSUMEL,SSUMKO,IU,IPLOT,ITITLE,TRETRN,``` | $\begin{array}{ll}\text { PT } \\ \text { PI } & 10 \\ 20\end{array}$ |
| :---: | :---: |
| C********************************************************************** | P1 30 |
|  | el 00 |
| C THIS ROUTINE, BUBROUTINE OF THE MAIN PROGRAM PERYORM, PLOT8 WALL | PY 50 |
| C PRESSURE OIFFERENTIAL ANOIOR CUMMULATIVE, NONOIMENSIONAL PRESSURE | P1 60 |
| C LOSSES against cummulative circuit centenline lengit. this plot | P1 70 |
| c SUgROUTINE WA WRITTEN POR - LETA PLOTTER WITH O.OOSEINCH INCREMENTS. | PL 60 |
| C NOPE... WHEN PLOTYING IN SI UNIT8, CENTIMETER SCALES WILL RESULT'. | 0 |
|  | 17 |
| C********************************************************************** | -1 110 |
| OIMENSION OEP P (32).171TLE(21),38UMEL (32),88UMKO(32) | If 120 |
| DIMENSION IX (6),IXN(6), IXNM(6),IY(6),IYN(6),IYNM(6) | -1 130 |
| coene. | ei 140 |
| PLOT AXIS LABELS ARRAYS | PT 150 |
|  | P) 160 |
| DAFA IXN(1),IXN(2),IXN(3),IXN(4), IXN(5),IXN(6)/ 4HCIRC,4HUIT | PI 190 |
| 1 GHLENG, 4 HTH (CAHFEET, 4 H) , | PI 180 |
| DATA IXNM (1), IXNM (2), IXNM (3), IXNM (4), IXNM (5),IXNM (6)/ 4HCIRC, | PT 190 |
|  | PI 200 |
|  | PT 210 |
|  | PI 220 |
| DATA IYNM(1),IYNM(2),IYNM(3),IYNM (4) OIYNM(5),IYNM(6)/ UHWALL, | pr 230 |
| 1: $4 H$ PRE $4 \mathrm{HSSUR,4HE} \mathrm{(Ne4H/SQ}$,4 HM ) , | PI 240 |
|  | p1 250 pi 200 |
| CaREREADYI ${ }^{\text {c }}$ G Of THE PLOTTER AND ESTABLISHMENT OF PME ORIGIN |  |
|  | P1 <br> pf 270 <br> 180 |
|  | pi 290 |
| CaBCo DEFINITION OF PLOTIER PARAMETERS IN SI | P1 <br> 10 |
|  | Q1 320 |
| $\mathrm{N}^{2}-\mathrm{N}+2$ | PT 330 |
| FACT 2. | RI 340 |
| XLEN 15. | QT 350 |
| YLEN $10^{\circ}$ | PT 360 |
| PYLEN YLEN-gS | PT 370 |
| YLAE Y YLEN+ 1 | PT 380 |
| XNEXT = 17. | Pt 390 |
| Ymax $=11$. | 21. 400 |
| YMARG $=.5$ | PT 4:0 |
| Cosese. | P1 420 |
| C.0. DEFINITION OF AXIS LABELS | PT 430 |
|  | PI 440 |
| IY(1):IYN(1) | PI 460 |
| $I \times(I)=I \times N(I)$ <br> continue | Pi 470 |
|  | -1 490 |
| OCACiĖEİEROR TYPE OE UNITS | PY 1.500 |



```
    CAbL PLOF(0,0,=YMAX,=3)
    PT }101
    CALL PLOT(O,OPYMARGRE3)
    CALL PLOT(XNEXT,O,0,03)
C._(OTON_,_
    C.0.01900.0
Cg-gelF NO MORE CISES, CONTRDL PERMANENILY IAKEN PROM PLOITER
D] }102
    PT }103
er b040
C...
```


$\qquad$

```
    RETURN
    pi }105
EI 1000
pT }107
OL1080
ConocosARANETER CONVERSION TO 8I UNITS
C300O CONTINUE
    FACT : 2.15.26999
    XLEN=15,$1.26999
    YLEN e 10;el:26990
    YLAB M YEN+,I
    XNEXT =11,1,20999 _OI 1180
    YMAX = 11.$1,26999
YMARG 5%1 26999
PT1190
    O1 1200
pi 1210
    003500 la I:6
    3500 IY(I) : IXNM(I)
    |1 }122
    P9 1230
    00 10 1000
    END
ei 12:0
p! }125
```

APPENDIX D

INPUT AND OUTPUT FOR SAMPLE CASES

Six wind tunnels were used, in addition to the test case (fig. 11), as . sample cases to establish the reliability and accuracy of the computer program analysis technique for the various types of duct components and wind tunnel circuits. Each case included here is titled with the appropriate wind tunnel name and its pages are numbered. The performance analyses are presented on the first two to three pages of each case. The summary characteristics tabulations and the plotted information were omitted. The annotated tabulations of the input data were included for reference.

The results of the performance analyses are summarized in table 6. They are discussed and critiqued in the Results and Evaluation sections of this report.

$$
i_{i}^{*}
$$



| NASA-AMES |  |  |  |  |  |  |  |  |  |  |  | $\begin{aligned} & \text { PAGE } 2 \\ & \text { DP } / 00 \end{aligned}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| NQ. SECTION TYPE | SHAPE | H1 | W1, D1 | AREAI | A1/ 10 | $A R, C R$ | 2 THETA | $V 1$ | MACH1 | LENGTH | DP/QL |  |
| +4 +----------4 | ---+ | $\begin{aligned} & \text { H2 } \\ & \text { METERS } \end{aligned}$ | W2,02 METERS $\qquad$ | $\begin{aligned} & \text { AREAZ } \\ & \text { Sin } \\ & +--\infty \end{aligned}$ | $42 / 40$ $+--\infty+$ |  | $\begin{aligned} & \text { DEGREFS } \\ & +-\infty+ \end{aligned}$ | $\begin{gathered} \mathrm{V} 2 \\ M / S E C \\ +---\infty \end{gathered}$ | MACH2 <br> +ー-- + | METERS | +---4 | +---+ |
| 10 DIFFUSEP. | $\begin{aligned} & \text { RECT } \\ & \text { CIRC } \end{aligned}$ | 5.68 | $\begin{aligned} & 6.60 \\ & 8.71 \end{aligned}$ | $\begin{array}{r} 37.47 \\ 59.56 \\ \hline \end{array}$ | $\begin{aligned} & 5.76 \\ & 9.16 \\ & \hline \end{aligned}$ | 1.59 | 6.01 | $\begin{array}{r} 21.4 \\ 13.5 \\ \hline \end{array}$ | $\begin{aligned} & 0.063 \\ & 0.040 \\ & \hline \end{aligned}$ | 17.16 | 0.02364 | 0.00066 |
| 11 FAN COATRACTIUN | $\begin{aligned} & \text { CIRC } \\ & \text { CIRC } \\ & \hline \end{aligned}$ |  | $\begin{array}{r} 8.71 \\ 8.87 \\ \hline \end{array}$ | $\begin{aligned} & 59.56 \\ & 46.04 \\ & \hline \end{aligned}$ | $\begin{aligned} & 9.16 \\ & 7.68 \\ & \hline \end{aligned}$ | 1.29 | 38.48 | $\begin{array}{r} 13.5 \\ 17.4 \\ \hline \end{array}$ | $\begin{array}{r} 0.040 \\ 0.051 \\ \hline \end{array}$ | 1.51 | 0.00066 | 0.00001 |
| 12 FAN OUCT \& STKUTS | $\begin{aligned} & \text { CIRC } \\ & \text { CIRC } \end{aligned}$ |  | $\begin{aligned} & 8.87 \\ & 8.87 \end{aligned}$ | $\begin{array}{r} 46.04 \\ 46.04 \end{array}$ | $\begin{aligned} & 7.08 \\ & 7.08 \\ & \hline \end{aligned}$ |  |  | $\begin{array}{r} 17.4 \\ 17.4 \\ \hline \end{array}$ | $\begin{aligned} & 0.051 \\ & 0.051 \\ & \hline \end{aligned}$ | 0.61 | 0.00914 | 0.00017 |
| 13 MULT INTRNL STRCTR | $\begin{aligned} & \text { CIRC } \\ & \text { CIRC } \end{aligned}$ |  | $\begin{array}{r} 8.87 \\ 8.87 \\ \hline \end{array}$ | $\begin{array}{r} 46.13 \\ 61.75 \\ \hline \end{array}$ | $\begin{array}{r} 7.09 \\ 9.49 \\ \hline \end{array}$ |  |  | $\begin{array}{r} 17.4 \\ 13.0 \\ \hline \end{array}$ | $\begin{aligned} & 0.051 \\ & 0.038 \\ & \hline \end{aligned}$ |  | 0.01129 | 0.00021 |
| 14 FAN DIFSRECATR BDY | $\begin{aligned} & \text { CIRC } \\ & \text { RECT } \end{aligned}$ | 9.14 | $\begin{array}{r} 8.87 \\ 10.06 \\ \hline \end{array}$ | $\begin{aligned} & 55.84 \\ & 91.99 \end{aligned}$ | $\begin{array}{r} 8.59 \\ 14.14 \\ \hline \end{array}$ | 1.65 | 7.54 | $\begin{array}{r} 14.3 \\ 8.7 \\ \hline \end{array}$ | $\begin{aligned} & 0.042 \\ & 0.026 \\ & \hline \end{aligned}$ | 18.14 | 0.05617 | 0.00071 |
| 15 CONTRACTN, SINGLE | $\begin{aligned} & \text { RECT } \\ & \text { RECT } \end{aligned}$ | $\begin{array}{r} 9.14 \\ 8.19 \\ \hline \end{array}$ | $\begin{array}{r} 10.06 \\ 9.11 \\ \hline \end{array}$ | $\begin{array}{r} 91.99 \\ 74.61 \\ \hline \end{array}$ | $\begin{array}{r} 14.14 \\ 11.47 \\ \hline \end{array}$ | 1.23 | 11.51 | $\begin{array}{r} 8.7 \\ 10.7 \\ \hline \end{array}$ | $\begin{aligned} & 0.026 \\ & 0.032 \\ & \hline \end{aligned}$ | 5.33 | 0.00171 | 0.00001 |
| 16 DICFUSEF | $\begin{aligned} & \text { RECT } \\ & \text { RECT } \end{aligned}$ | $\begin{array}{r} 8.19 \\ 8.52 \\ \hline \end{array}$ | $\begin{array}{r} 9.11 \\ 9.43 \\ \hline \end{array}$ | $\begin{array}{r} 74.61 \\ 80.37 \\ \hline \end{array}$ | $\begin{array}{r} 11.47 \\ 12.36 \\ \hline \end{array}$ | 1.03 | 6.93 | $\begin{array}{r} 10.7 \\ 10.0 \\ \hline \end{array}$ | $\begin{aligned} & 0.032 \\ & 0.029 \\ & \hline \end{aligned}$ | 3.05 | 0.00306 | 0.00002 |
| 17 DIFFUSEP | $\begin{aligned} & \text { RECT } \\ & \text { RECT } \end{aligned}$ | $\begin{array}{r} 8.52 \\ 9.14 \\ \hline \end{array}$ | $\begin{array}{r} 9.43 \\ 10.06 \\ \hline \end{array}$ | $\begin{array}{r} 80.37 \\ 91.99 \\ \hline \end{array}$ | $\begin{array}{r} 12.36 \\ 14.14 \\ \hline \end{array}$ | 1.14 | 7.58 | $\begin{array}{r} 10.0 \\ 8.7 \\ \hline \end{array}$ | $\begin{aligned} & 0.029 \\ & 0.026 \\ & \hline \end{aligned}$ | 5.33 | 0.00565 | 0.00003 |
| 16 CONSTANT AREA DUCT | $\begin{aligned} & \text { RECT } \\ & \text { RECT } \end{aligned}$ | $\begin{array}{r} 9.14 \\ 9.14 \\ \hline \end{array}$ | $\begin{aligned} & 10.06 \\ & 10.06 \\ & \hline \end{aligned}$ | $\begin{array}{r} 91.99 \\ 91.99 \\ \hline \end{array}$ | $\begin{aligned} & 14.14 \\ & 14.14 \end{aligned}$ |  |  | $\begin{aligned} & 8.7 \\ & 8.7 \end{aligned}$ | $\begin{aligned} & 0.026 \\ & 0.026 \\ & \hline \end{aligned}$ | 1.52 | 0.00140 | 0.00001 |
| 19 CCRAFR WITH VANES | $\begin{aligned} & \text { RECT } \\ & \text { RECT } \end{aligned}$ | $\begin{aligned} & 9.14 \\ & 9.14 \\ & \hline \end{aligned}$ | $\begin{aligned} & 10.06 \\ & 10.06 \\ & \hline \end{aligned}$ | $\begin{aligned} & 91.99 \\ & 91.99 \\ & \hline \end{aligned}$ | $\begin{aligned} & 14.14 \\ & 14.14 \\ & \hline \end{aligned}$ |  |  | $\begin{aligned} & 8.7 \\ & 8.7 \\ & \hline \end{aligned}$ | $\begin{array}{r} 0.026 \\ 0.026 \\ \hline \end{array}$ | 10.52 | 0.17085 | 0.00079 |
| 20 COASTANT AREA DUCT | $\begin{aligned} & \text { RECT } \\ & \text { RECT } \end{aligned}$ | $\begin{array}{r} 9.14 \\ 9.14 \\ \hline \end{array}$ | $\begin{aligned} & 10.06 \\ & 10.06 \\ & \hline \end{aligned}$ | $\begin{array}{r} 91.99 \\ 91.90 \\ \hline \end{array}$ | $\begin{aligned} & 14.14 \\ & 14.14 \\ & \hline \end{aligned}$ |  |  | $\begin{aligned} & 8.7 \\ & 8.7 \end{aligned}$ | $\begin{array}{r} 0.026 \\ 0.026 \\ \hline \end{array}$ | 5.38 | 0.00494 | 0.00002 |
| 21 CORAER WITH VAIVES | $\begin{aligned} & R E C T \\ & R E C T \end{aligned}$ | $\begin{aligned} & 9.14 \\ & 9.14 \\ & \hline \end{aligned}$ | $\begin{aligned} & 10.06 \\ & 10.06 \\ & \hline \end{aligned}$ | $\begin{array}{r} 91.99 \\ 91.99 \\ \hline \end{array}$ | $\begin{array}{r} 14.14 \\ 14.14 \\ \hline \end{array}$ |  |  | $\begin{aligned} & 8.7 \\ & 8.7 \end{aligned}$ | $\begin{aligned} & 0.026 \\ & 0.026 \\ & \hline \end{aligned}$ | 10.29 | 0.18071 | 0.00084 |

PAGE 3

PAGE 4




** '* NASA-AMES RESEARCH CENTER 7- BY 10-FOOT WIND TUNNEL *' CASE COMPLETED OR TERMINATED. **
SINGLE-RETURN, CLOSEC-TEST-SECTION WIND-TUNNEL PERFORMANCE
ATMOSPFEPIC PRESSUKE $=1.000$ ATMOSPHERES $=101325.0 \mathrm{~N} / S Q \mathrm{M}$
TEST SFCTICN CLNDITIUNS E-

$$
\text { MO. SECTICNTYFL SHAPE HI W1,D1 AREAI A1/AO AR ,CR } \angle \text { THETA VI MACHI LENGTH DP/QL DP/QO }
$$

$$
10 \equiv \star
$$

$$
2 \text { CONTRACTN, SINGLE RECT }
$$

$$
4 \text { CONTRACTN, SINGLE }
$$


5 DIFFUSFR
6 CIFFUSER
8 CIFFUSFR

$$
\begin{aligned}
& R E C T \\
& R=C T
\end{aligned}
$$

RECT

$$
\begin{aligned}
& R E C T \\
& R E C T
\end{aligned}
$$

$$
3 \text { TFST SECT, CIFSH }
$$

7 CORNER WITH VAATHS
9 COPAEF WITH VANES

$$
\begin{aligned}
& \text { RECT } \\
& \text { RECT }
\end{aligned}
$$

$$
\begin{aligned}
& R \equiv C T \\
& R \equiv C T
\end{aligned}
$$

$$
\begin{aligned}
& 15.70 \\
& 9.14 \\
& 7.14 \\
& 9.14 \\
& 0.14 \\
& 4.95 \\
& 4.95 \\
& 4.95 \\
& 4.95 \\
& 8.69 \\
& 8.69
\end{aligned}
$$

$$
\begin{aligned}
& 8.69 \\
& 8.69
\end{aligned}
$$

$$
\begin{aligned}
& 3.69 \\
& 9.15
\end{aligned}
$$

$$
\begin{aligned}
& 9.75 \\
& 0.75
\end{aligned}
$$

$$
\begin{array}{r}
15.70 \\
7.92
\end{array}
$$

$$
7.92
$$

$$
\begin{array}{rrrrrrrrr}
8.12 & 74.22 & 1.02 & 2.11 & 24.44 & 51.0 & 0.150 & 7.01 & \\
\hline 7.09 & 35.10 & 0.48 & & & 112.7 & 0.335 & & 0.00254 \\
\hline
\end{array}
$$

$$
\begin{array}{llllllllll}
7.09 & 35.10 & 0.48 & 1.02 & 0.26 & 112.7 & 0.335 & 13.11 & 0.01483 & 0.06602 \\
\hline
\end{array}
$$

$$
\begin{array}{lllllllll}
35.74 & 0.49 & 2.11 & 5.87 & 110.5 & 0.328 & 29.79 & 0.06272 & 0.26884 \\
\hline 75.46 & 1.04 & & & 50.1 & 0.148 & & &
\end{array}
$$

$$
\begin{array}{lllllll}
75.46 & 1.04 & 50.1 & 0.148 & 9.75 & 0.14222 & 0.13102 \\
75.46 & 1.04 & 50.1 & 0.148 & & &
\end{array}
$$

$$
\begin{array}{lllllll}
75.46 & 1.04 & 50.1 & 0.148 & 9.75 & 0.14222 & 0.13102 \\
\hline
\end{array}
$$

$$
8.69
$$

$$
\begin{array}{lll}
9.75 & 95.14 \\
9.75 & 95.14 & \frac{1.31}{1.31}
\end{array}
$$

$$
\begin{array}{lllll}
39.6 & 0.117 & 10.82 & 0.14366 & 0.08292 \\
\hline
\end{array}
$$

$$
\begin{array}{ll}
39.0 & 0.111 \\
\hline 9.6 & 0.117
\end{array}
$$

$$
\begin{gathered}
-5 \\
x 4 \\
x
\end{gathered}
$$

LOCKHEEL-GEORGIA LDW-SPEED WIND TUNNEL, V/STOL TEST SECTIUN * * ...CUNTINULD.... $\quad$ PAGE 2
NO. SECTION TYPE SHAPE H1 W1,D1 AREAI AI/AO AR,CR 2 THETA VI MACHI LENGTH DP/QL DP/QD METERS

$+\cdots+\cdots+\cdots+\cdots$ | 39.6 | 0.117 | 0.06777 | 0.03912 |
| :--- | :--- | :--- | :--- |
| 39.6 | 0.117 |  |  |

$3.83 \quad 0.00327 \quad 0.00189$
$\begin{array}{llllllll}105.32 & 1.45 & 1.51 & 15.75 & 35.7 & 0.105 & 7.83\end{array}$
$69.62 \quad 0.96 \quad \begin{array}{lllll} & 54.5 & 0.161 & 0.00244 & 0.00265\end{array}$
$\begin{array}{lllllll}69.62 & 0.96 & 54.5 & 0.161 & 5.64 & 0.02219 & 0.02407 \\ 69.02 & 0.96 & 54.5 & 0.101 & & & \end{array}$
$\begin{array}{rrrrrrrrr}69.62 & 0.96 & 1.57 & 11.18 & 54.5 & 0.161 & 12.65 & 0.02837 & 0.03077 \\ 111.03 & 1.53\end{array}$
$55.17 \quad 0.03921 \quad 0.01659$
$\begin{array}{lllll}15.2 & 0.045 & 17.53 & 0.14696 & 0.01257 \\ 15.2 & 0.045 & & & \end{array}$

$\begin{array}{lllll}15.2 & 0.045 & 17.53 & 0.14096 & 0.01257\end{array}$
1.960610 .16763
TOTAL CENTERLINE LENGTH $=240.87$ mETERS
PAGE 3



LOCKHEED-GEURGIA LOW-SPEED WIND TUNNEL, V/STO TEST SECTION $\quad$. 5



| PER- | 10(-6) PER- PER- |  |
| :--- | :--- | :--- | :--- | :--- |
| CENT DEG | METERS | CENT CENT |



## SECTION CESCRIPTION DATA

-EMPTY: INOICATES OPTIONAL, NSN-RE QUIRED INPUT PARAMETER HAS BEEN OMITTEO OR PARAMETER MAY BE INTENDED AS $2 E R E$ IE

PAGE 6.






LOCKHEED-GEORGIA LOW-SPEED WIND TUNNEL, V/STOL TEST SECTION *
** ** LUCKHEED-GEURGIA LOW-SPEED WINU TUNNEL, VTSTOL TEST SECTION * CASE COMPLETEU GR TERMINATED. *
PAGE 1
INDIAN INSTITUTE OF SCIENCE 14- BY 9-FGOT WIND TUNNEL AT BANGALORE
non-return, closec-test-section wind-tunnel performance

SHETION TYPE SHAPE HI WI,DI AREAI AI/AO AR,CR 2 THETA VI MACHI LENGTH DP/QL DP/QO AREAZ AZ/AO

$\begin{array}{lllllll}48 & 14.10 & 6.6 & 0.019 & 0.25621 & 0.00124\end{array}$ $\begin{array}{llll}148.62 & 14.10 & 6.6 & 0.019\end{array}$ $\begin{array}{lll}148.62 & 14.10 & 6.6 \quad 0.019\end{array}$ $6.6 \quad 0.019$ $6.6 \quad 0.019$
$0.25621 \quad 0.00124$

$0.25621 \quad 0.00124$ | 6.6 | 0.019 |  |  |  |
| :--- | :--- | :--- | :--- | :--- |
| 6.6 | 0.019 |  | 0.25621 | 0.00124 |
| 6.6 | 0.019 |  |  |  |
| 6.6 | 0.019 | 17.86 |  |  |


| 6.6 | 0.019 | 17.86 |  |  |
| :--- | :--- | :--- | :--- | :--- |
| 54.3 | 0.160 |  | 0.00941 | 0.00307 |


| 54.3 | 0.260 | 5.49 |  |  |
| :--- | :--- | :--- | :--- | :--- |
| 0.00357 | 0.00357 |  |  |  |

$0.10 \quad 0.01241 \quad 0.01241$
$\begin{array}{llllllllllllllll}11.33 & 1 . C 7 & 3.67 & 5.40 & 89.1 & 0.264 & 36.85 & 0.08077 & 0.06953\end{array}$ $\begin{array}{llll}11.33 & \frac{1.07}{31.53} & 23.5 & 0.069\end{array}$
1.53 3.4


MAXIMUM VELOCITY FOR A SPECIFIED POWER CONSUMPTION
THE MAXIMUM TEST SECTICN FLOW ACHIEVABLE WITH 862800. WATTS OF POWER AVAILABLE IS APPROXIMATELY AS FOLLOWS --

INDIAN INSTITUTE OF SCIENCE 14- BY 9-FOOT WIMD TUWNEL AT BANGALORE



IALIAN INSTITUTE OF SCIENCE 14- BY 9-FUOT WIND TUNNEL AT BANGALQRE *_...CONTINUED.... *__ PAGE 5
** * IADIAN INSTITUTE OF SCIENCE 14- by g-foot wind tunnel at bangalore ** case completed or terminated. **

PA 2

—

PAGE 4
HAWKER SICDELEY AIRCRAFT 15-FDOT V/STOL WIND TUNNEL AT HATFIELD

CASE TERMINATION CONDITIONS DATA



PAGE 6
 ** 1*

UNIVERSITY OF WASHINGTON 8-BY 12-FOOT WIND TUNNEL_ 2



 \begin{tabular}{ccccccccccc|}
5.16 \& 29.8 \& 0.088 \& 6.40 \& 0.01563

 $23.3 \quad 0.069 \quad 10.59 \quad 0.02002 \quad 0.00084$ 

\& 0.069 \& 10.59 \& 0.02002 \& <br>
\hline
\end{tabular} $17.6 \quad 0.052 \quad 0.76 \quad 0.00129 \quad 0.00003$

$17.6 \quad 0.052 \quad 4.50 \quad 0.11603 \quad 0.00277$ $\begin{array}{lllll}17.6 & 0.052 & 4.50 & 0.11603 & 0.0027 \\ 17.6 & 0.052 & & \end{array}$


[^2]...COntinued.... Page 3

UNIVERSITY OF WASHINGTON 8- BY 12-FOCT WIND TUNNEL

| ANNOTATED INPUT DATA TABULATION |
| :--- |
| 'EMPTY' INOICATES OPTIONAL, NCN-REGUIRED INPUT PARAMETER HAS BEEN OMITTEO OR PARAMETER MAY BE INTENOED AS ZERO. |
| 'ERROR INCICATES MANOATORY INPUT PARAMETER HAS BEEN OMITTED. THIS MUST BE CORRECTED BEFORE COMPUTATION IS POSSIBLE. |
| 'EXTRA' INOICATES SUPERFLUOUS INPUT PARAMETER HAS BEEN UNNECESSARILY INCLUDED ON INPUT CARD AND MAY BE REMOVED. |
| -OPT'A' INDICATES OPTIONAL INPUT DATA HAS BEEN OMITTED AND THE PARAMETER WILL DEFAULT TO A PREDETERMINED VALUE. |



ANNCTATED INPUT DATA TABULATION

[^3]
UNIVERSITY OF WASHINGTON 8- BY 12-FOOT WIND TUNNEL

[^4]ATMOSPHERIC PRESSURE $=1.000$ ATMOSPHERES $=101325.0$ N/SO M.
TEST SECTICN CCNDITIONS --
TOTAL PRESSURE $=1.000$ ATMOSPHERES $=101325.0 \mathrm{~N} /$ SQ M.
NO. SECTION TYPE SHAPE HI WI,DI AREAL AI/AO AR,CR 2 THETA VI MACHI LENGTH DP/QL DP/QO
METERS METERS SOM METERS M/SEC
$0.00382 \quad 0.00382$


$47.7 \quad 0.141 \quad 0.00073 \quad 0.00060$

| 4 DIFFUSER | FL 0 | 9.60 | 19.20 | 164.56 | 1.10 | 1.16 | 12.61 | 47.7 | 0.141 | 5.18 | 0.01578 | 0.01296 |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- |
|  | FL O | 10.21 | 20.96 | 191.63 | 1.28 |  |  | 40.9 | 0.120 |  |  |  |
| 5 FAN CONTRACTION | FL 0 | 10.21 | 11.57 | 191.52 | 1.28 | 1.22 | 6.35 | 40.9 | 0.120 | 9.45 |  |  |

$50.10 .148 \quad 0.002850 .00258$
$\begin{array}{lllll}50.1 & 0.148 & 3.81 & 0.01708 & 0.01545\end{array}$
$\begin{array}{llllll}160.02 & 1.07 & 1.25 & 11.74 & 49.1 & 0.145\end{array}$
$\begin{array}{lllll}49.1 & 0.145 & 5.77 & 0.02019 & 0.0088\end{array}$
$\begin{array}{lllll}39.2 & 0.115 & 31.55 & 0.03618 & 0.02008\end{array}$
$\begin{array}{lllllllll}393.12 & 2.63 & 1.04 & 1.79 & 19.8 & 0.058 & 15.39 & 0.10037 & 0.01433\end{array}$ $\begin{array}{ll}393.12 & 2.63 \\ 410.23 & 2.75\end{array}$

 | 2.38 | $\begin{array}{lllll}11.1 \\ 10.6 & 0.033 \\ 0.031\end{array}$ | 16.92 | 0.09771 | 0.00436 |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- |
| TOTAL CENTERLINE LENGTH $=$ |  |  |  |  |


PAGE 3

ANNCTATED INPUT DATA TABULATION
'EMPTY' INCICATES DPTIONAL, NON-REQUIRED INPUT PARAMETER HAS BEEN OMITTFD CR PARAMETER MAY BE INTENDED AS ZERC.
FRRQR' INDICATES MANDATORY INPUT PARAMETER HAS BEEN OMITTED. THIS MUST BF CORRECTED BEFORE COMPUTATION IS POSSIBLE.
'EXTRA' INCICATES SUPERFLUOUS INPUT PARAMETER HAS BEEN UNNECESSARILY INCLUDED ON INPUT CARD AND MAY BE REMOVED.
'OPT'A' INDICATES OPTIONAL INPUT DATA HAS BEEN OMITTED AND THE PARAMETER WILL DEFAULT TO A PREDETERMINED VALUE.

PGE 4
NASA-LANGLEY RESEARCH CENTER 30- BY G0-FOOT WIND TUNNEL


PAGE 5

NASA-LANGLEY RESEARCH CENTER 30- BY 60 -FOOT WIND TUNNEL * CASE COMPLETED OR TERMINATED. *:
7JNNnI ONIM LOOJ-09 A8 $-0 \varepsilon$ \&
$\cdots \cdot \square$

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TABLE 1.- NUMERIC INPUT CODE DEFINITIONS

| Code type | Code value | Description of code meaning |
| :---: | :---: | :---: |
| Tunnel type | $\begin{aligned} & 1 \\ & 2 \\ & 3 \\ & 4 \\ & 5 \\ & 6 \end{aligned}$ | Closed test section, single-return tunnel Closed test section, double-return tunnel Closed test section, non-return tunnel Open-throat, single-return tunnel Open-throat, double-return tunnel Open-throat, non-return tunnel |
| Units of measure $\downarrow$ Section shape | 1 | International System of Units (SI) |
|  | 2 | U.S. Customary Units |
|  | 1 | Circular cross section |
|  | 2 | Rectangular cross section |
| Section type | 3 | Flat oval cross section (ceiling and floor parallel with semicircular sidewalls) (See table 4) |
| Section type <br> Plot type | $\leq 0.0$ | No plots |
|  | 1.0 | Cummulative pressure losses vs circuit length |
|  | 2.0 | Wall pressure differential vs circuit length |
|  | >2.0 | Cummulative pressure losses and wall pressure differential vs circuit length (on separate plots) |

TABLE 2.- TUNNEL MASTER CONTROL INPUT DATA DESCRIPTIONS

| $\begin{gathered} \text { Card } \\ \text { column }(s) \end{gathered}$ | $\begin{aligned} & \text { Field } \\ & \text { title(s) } \end{aligned}$ | Requirement? ${ }^{\text {a }}$ | Input type | Description(s) | Units |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 1 |  | Required | Minus sign | Master card identifier | - |
| 2 | CASE SEQ. | \{optional | Integer | Arbitrary user case number | --- |
| 3 | TUNNEL TYPE | Optional | Integer | Tunnel type code (see table 1) | --- |
| 4 | UNITS | Default (1) | Integer | Units of measure code (see table 1) | - |
| 5 | SECT. INLET SHAPE | Required | Integer | Test section upstream end shape code (see table 1) | --- |
| 6 | SECT. EXIT SHAPE | Optional | Integer | Test section downstream end shape code (see table 1) | --- |
| 11-15 | H1 | Geom. Dep. | Real | Height of rectangular or flat oval test section at upstream end | $m$ or ft |
| 16-20 | W1, D1 | Required | Real | Width of rectangular or flat oval, or diameter of circular test section at upstream end | m or ft |
| 21-25 | MODEL BLKGE | Optional | Real | ```Blockage factor of the model in the test section (if model is to be included)``` | \% of <br> test <br> section <br> area |
| 26-30 | V0 | Required | Real | Test section velocity for which power calculation is to be made | $\mathrm{m} / \mathrm{sec}$ or |
| 31-35 | POWER LEVEL | Optional | Real | Power for which maximum attainable velocity is to be calculated (if velocity-optimizing is requested) | ft/sec <br> $10^{6} \mathrm{~W}$ <br> or <br> $10^{3} \mathrm{hp}$ |
| 36-40 | PT | Default(1.0) | Rea1 | Test section total (stagnation) pressure | ATM |
| 41-45 | TT | Required | Real | Test section total (stagnation) temperature | ${ }^{\circ} \mathrm{C} \text { or }$ ${ }^{\circ} \mathrm{F}$ |
| 46-50 | P ATM | Default (1.0) | Real | External atmospheric pressure | ATM |

[^5]TABLE 3.- SECTION INPUT DATA DESCRIPTIONS

| $\begin{gathered} \text { Card } \\ \text { column }(s) \end{gathered}$ | $\begin{aligned} & \text { Field } \\ & \text { title(s) } \end{aligned}$ | Requirement ${ }^{\text {a }}$ | Input type | Description(s) | Units |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 1-2 | SECT. SEQ. | Optional | Integer | Arbitrary section order number | --- |
| 3-4 | SECT. TYPE | Required | Integer | Section type code (see table 4) | --- |
| 5 | SECT. INLET SHAPE | Required | Integer | Section upstream end shape code (see table 1) | --- |
| 6 | SECT. EXIT SHAPE | Required | Integer | Section downstream end shape code (see table 1) | --- |
| 7-8 | TOTAL NO. DUCTS | Default (1.0) | Rea1 | Number of multiple ducts | --- |
| 9-10 | ITEMS PER DUCT | Default (1.0) | Real | ```Number of individual, flow obstruction, drag loss items in local duct``` | --- |
| 11-15 | H1 | Geom. Dep. | Real | Height of upstream end of noncircular section | $m$ or ft |
| 16-20 | W1, D1 | Required | Real | Width of non-circular, or diameter of circular section at upstream end | m or ft |
| 21-25 | L | Sect. Dep. | Rea1 | Centerline length of section | $m$ or ft |
| 26-30 | H2 | Geom. Dep. | Real | Height of downstream end of noncircular section | $m$ or ft |
| 31-35 | W2, D2 | Required | Real | Width of non-circular, or diameter of circular section at downstream end | $m$ or ft |
| 36-40 | $\begin{aligned} & \text { L/DH, } \\ & \text { S/AL } \end{aligned}$ | Sect. Dep. | Real | Length-to-hydraulic diameter, ratio of flow straightener cells, or drag-area-to-local-duct-flow-area ratio for each flow obstruction drag item | $\mathrm{m} / \mathrm{m}$ or $\mathrm{ft} / \mathrm{ft}$, $\mathrm{m}^{2} / \mathrm{m}^{2}$ <br> or $\mathrm{ft} \mathrm{t}^{2} / \mathrm{ft}^{2}$ |
| 41-45 | DHUB, CHORD, DMESH | Sect. Dep. | Real | Hub diameter of fan-drive section, or turning vane chord length, or mesh screen wire diameter | m or ft |

TABLE 3.- SECTION INPUT DATA DESCRIPTIONS - Concluded.

| $\begin{gathered} \text { Card } \\ \text { column(s) } \end{gathered}$ | $\begin{aligned} & \text { Field } \\ & \text { title(s) } \end{aligned}$ | Requirement? ${ }^{\text {a }}$ | Input type | Description(s) | Units |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 46-50 | BLKGE PRSTY | Sect. Dep. | Rea1 | Local flow area blockage due to each obstruction in the local duct, or porosity of flow straighteners, screen, perforated plate | \% of local area |
| 51-55 | PHI | Sect. Dep. | Real | ```Corner flow centerline turning angle, 0``` | deg |
| 56-60 | KEXP <br> KMESH | $\begin{aligned} & \text { Default (INT) } \\ & \text { Default }(1.3) \end{aligned}$ | Real | Diffuser expansion loss parameter (see fig. 4), or | --- |
|  | KT 90 | $\left\{\begin{array}{l} \text { Default }(0.15) \\ \text { Default }(1.80) \end{array}\right.$ |  | mesh screen loss constant, or $\left\{\begin{array}{c}\text { turning vane loss parameter at } \\ \phi=90^{\circ} \\ \text { empty corner loss parameter at } \\ \phi=90^{\circ}\end{array}\right.$ |  |
| 61-65 | $\begin{aligned} & \mathrm{CD} \\ & \mathrm{~K} \end{aligned}$ | Sect. Dep. | Real | Drag coefficient of flow obstruction, or fixed, known local loss value | $10^{6}$ |
| 66-70 | RNREF, RUFNESS | $\begin{aligned} & \text { Default }(0.5) \\ & \text { Default }(.0001 \mathrm{~m}) \end{aligned}$ | Real | Reference Reynolds number for which $90^{\circ}$ corner loss value is given, or surface roughness of flow straightener material | $\begin{aligned} & 10^{6} \\ & 10^{-6} \mathrm{~m} \\ & \text { or } \\ & 10^{-6} \mathrm{ft} \end{aligned}$ |
| 71-75 | ETA | Default (100.0) | Real | Efficiency of fan drive system |  |
| 76-80 | D EPS | Sect. Dep. | Real | Additional (amount over 100\%) downstream influence factor for flow obstruction items | $\begin{aligned} & \% \text { over } \\ & 100 \% \end{aligned}$ |

[^6]TABLE 4.- ADDITIONAL, SECTION-DEPENDENT INPUT REQUIREMENTS

| Section |  | $\begin{aligned} & \text { Additional } \\ & \text { input } \\ & \text { title(s) } \end{aligned}$ | Requirement? ${ }^{\text {a }}$ | $\begin{aligned} & \text { Card } \\ & \text { column(s) } \end{aligned}$ |
| :---: | :---: | :---: | :---: | :---: |
| Type description | Type code |  |  |  |
| Single ducts: |  |  |  |  |
| Test section, closed, constant area, empty | 01 | --- |  |  |
| Test section, closed, constant area with model | 02 | S/AL | Required | 36-40 |
|  |  | BLKGE | Optional | 46-50 |
|  |  | CD | Required | 61-65 |
|  |  | D EPS | Optional | 76-80 |
| ```Test section, closed, diffusing, empty Test section, closed diffusing,with model``` | 03 | KEXP | Default | 56-60 |
|  | 04 | S/AL | Required | 36-40 |
|  |  | BLKGE | Optional | 46-50 |
|  |  | KEXP | Default | 56-60 |
|  |  | CD | Required | 61-65 |
|  |  | D EPS | Optional | 76-80 |
| ```Test section, open-throat, empty``` | 05 | --- |  |  |
| Test section, open-throat, with model | 06 | S/AL | Required | 36-40 |
|  |  | BLKGE | Optional | 46-50 |
|  |  | CD | Required | 61-65 |
|  |  | D EPS | Optional | 76-80 |
| Constant-area duct | 10 | --- |  |  |
| Contraction | 20 | --- |  |  |
| ```Corner, constant-area, turning vanes only``` | 30 | CHORD | Required | 41-45 |
|  |  | PHI | Required | 51-55 |
|  |  | KT 90 | Default | 56-60 |
|  |  | RNREF | Default | 66-70 |
| Corner, constant-area, with turning vanes and walls | 32 | CHORD | Required | 41-45 |
|  |  | PHI | Required | 51-55 |
|  |  | KT 90 | Default | 56-60 |
|  |  | RNREF | Default | 66-70 |
| Corner, constant-area, with walls and without turning vanes | 33 | PHI | Required | 51-55 |
|  |  | KT 90 | Default | 56-60 |
| Corner, diffusing, with turning vanes and walls | 34 | CHORD | Required | 41-45 |
|  |  | PHI | Required | 51-55 |
|  |  | KT 90 | Default | 56-60 |
|  |  | RNREF | Default | 66-70 |
| Diffuser | 40 | KEXP | Default | 56-60 |
| Exit kinetic energy from flow dump | 45 | - |  |  |
| Sudden expansion | 46 | --- |  |  |

TABLE 4:- ADDITIONAL, SECTION-DEPENDENT INPUT REQUIREMENTS - Continued.

| Section |  | ```Additional input title(s)``` | Requirement? ${ }^{\text {a }}$ | $\begin{gathered} \text { Card } \\ \text { column }(s) \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: |
| Type description | Type code |  |  |  |
| Flow straighteners, thin honeycomb | 51 | L/DH | Required | 36-40 |
|  |  | PRSTY | Required | 46-50 |
|  | 52 | RUFNESS | Default | 66-70 |
| Flow straighteners, thick airfoils |  | L/DH | Default | 36-40 |
|  |  | PRSTY | Required | 46-50 |
| ```Perforated plate with sharp- edged orifices Woven mesh screen``` | 53 | PRSTY | Required | 46-50 |
|  |  |  |  |  |
|  | 54 | DMESH | Required | 41-45 |
|  |  | PRSTY | Required | 46-50 |
|  |  | KMESH | Default | 56-60 |
| ```Internal structure (drag item(s)) at upstream end of section``` | 56 | ITEMS | Default | 9-10 |
|  |  | S/AL | Required | 36-40 |
|  |  | BLKGE | Optional | 46-50 |
|  |  | CD | Required | 61-65 |
|  |  | D EPS | Optional | 75-80 |
| Fixed, known local loss item at upstream end of section Multiple ducts: | 57 | K | Required | 61-65 |
|  |  |  |  |  |
| Constant-area ducts | 61 | DUCTS | Required | 7-8 |
| Contractions | 62 | DUCTS | Required | 7-8 |
| Corners, constant-area; turning vanes only | 70 | DUCTS | Required | 7-8 |
|  |  | CHORD | Required | 41-45 |
|  |  | PHI | Required | 51-55 |
|  |  | KT 90 | Default | 56-60 |
|  |  | RNREF | Default | 66-70 |
| Corners, constant-area, with turning vanes and only one side-wall each | 71 | DUCTS | Required | 7-8 |
|  |  | CHORD | Required | 41-45 |
|  |  | PHI | Required | 51-55 |
|  |  | KT 90 | Default | 56-60 |
|  |  | RNREF | Default | 66-70 |
| Corners, constant-area, with turning vanes and walls | 72 | DUCTS | Required | 7-8 |
|  |  | CHORD | Required | 41-45 |
|  |  | PHI | Required | 51-55 |
|  |  | KT 90 | Default | 56-60 |
|  |  | RNREF | Default | 66-70 |
| Corners, constant-area, with walls and without turning vanes <br> Corners, diffusing, with turning vanes and only one side-wall each | 73 | DUCTS | Required | 7-8 |
|  |  | PHI | Required | 51-55 |
|  |  | KT $\because 0$ | Default | 56-60 |
|  | 74 | DUCTS | Required | 7-8 |
|  |  | CHORD | Required | 41-45 |
|  |  | PHI | Required | 51-55 |
|  |  | KT 90 | Default | 56-60 |
|  |  | RNREF | Default | 66-70 |

TABLE 4.- ADDITIONAL, SECTION-DEPENDENT INPUT REQUIREMENTS - Concluded.

| Section |  | Additional input title(s) | Requirement? ${ }^{\text {a }}$ | $\begin{gathered} \text { Card } \\ \text { column(s) } \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: |
| Type description | Type code |  |  |  |
| Corners, diffusing, with turning vanes and walls | 75 | DUCTS | Required | 7-8 |
|  |  | CHORD | Required | 41-45 |
|  |  | PHI | Required | 51-55 |
|  |  | KT 90 | Default | 56-60 |
|  |  | RNREF | Default | 66-70 |
| Diffusers | 84 | DUCTS | Required | 7-8 |
|  |  | KEXP | Default | 56-60 |
| Vaned diffuser | 85 | --- |  |  |
| Sudden expansion from multiple ducts to single duct | 86 | DUCTS | Required | 7-8 |
| Sudden expansion from multiple ducts to multiple ducts | 87 | Ducts | Required | 7-8 |
| Fan, constant-area annular duct(s) with motor-support strut(s) | 91 | DUCTS | Default | 7-8 |
|  |  | ITEMS | Default | 9-10 |
|  |  | S/AL | Required | 36-40 |
|  |  | DHUB | Required | 41-45 |
|  |  | BLKGE | Optional | 46-50 |
|  |  | CD | Required | 61-65 |
|  |  | ETA | Default | 71-75 |
|  |  | D EPS | Optional | 75-80 |
| Fan contraction(s) to annular duct(s) with motor-support strut(s) | 92 | DUCTS | Default | 7-8 |
|  |  | ITEMS | Default | 9-10 |
|  |  | DHUB | Required | 41-45 |
|  |  | BLKGE | Optional | 46-50 |
| Fan diffuser(s) from annular duct(s), each with tapering, cone-shaped centerbody | 94 | DUCTS | Default | 7-8 |
|  |  | DHUB | Required | 41-45 |
|  |  | BLKGE | Optional | 46-50 |
|  |  | KEXP | Default | 56-60 |
| Internal structure (drag item(s)) at upstream end of each duct | 96 | DUCTS | Required | 7-8 |
|  |  | ITEMS | Default | 9-10 |
|  |  | S/AL | Required | 36-40 |
|  |  | BLKGE | Optional | 46-50 |
|  |  | CD | Required | 61-65 |
|  |  | D EPS | Optional | 75-80 |
| Fixed, known local loss item at upstream end of each duct | 97 | DUCTS | Required | 7-8 |
|  |  | K | Required | 61-65 |

$\mathrm{a}^{\prime}$ "Default" indicates the input is optional and has a default value if omitted (see table 3).
"Optional" indicates the input may be selected and included as desired.
"Required" indicates the input must be non-zero and included for all sections of the specified type or the section will be skipped and the case not completed due to input error.

TABLE 5.- CASE TERMINATION TASK DESCRIPTIONS

| $\begin{aligned} & \text { Card } \\ & \text { column(s) } \end{aligned}$ | Input type | Input value | Task description |
| :---: | :---: | :---: | :---: |
| $\begin{gathered} 3-4 \\ 6 \end{gathered}$ | Blanks <br> Integer | B1anks | ```Case termination card identification Summary characteristics page(s): Non-print Print``` |
|  |  |  |  |
|  |  | $\underset{ }{0}$ |  |
| 7-8 | Real |  | Plotting of summary information as a function of distance through circuit: |
|  |  | $\leq 0.0$ | No plots |
|  |  | 1. | Cummulative pressure loss Wall pressure differential |
|  |  | $>2$. | Cummulative pressure loss and wall pressure differential |
| 9-10 | Real |  | Complete, annotated tabulation of input values: |
|  |  | 0.0 | No print unless internally forced by omission of required inputs |
|  |  | $\neq 0.0$ | "Chosen" tabulation |
| 11-15 | Real |  | Power-matching (optimizing velocity for a specified power level): |
|  |  | 0.0 | No velocity optimization |
|  |  | $\neq 0.0$ | Velocity optimization |
| 16-20 | Real |  | Return to beginning for evaluation of another case: |
|  |  | 0.0 | No return, program termination |
|  |  | $\neq 0.0$ |  |

TABLE 6.- COMPARISON OF PREDICTED WITH ACTUAL PERFORMANCE LEVELS FOR SEVERAL EXISTING WIND TUNNEL FACILITIES

| Wind tunnel description |  |  |  | Reference condition | Basis of actual energy ratio |  |  |  | Energy ratio |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Facility | Return circuit type/basic cross-section shape | Test section type/shape | Comments | ```Test section velocity, m/sec``` | Drive power |  |  | $\begin{aligned} & \text { Circuit } \\ & \text { losses } \end{aligned}$ | Actual ${ }^{\text {a }}$ | Estimated by computer program | $\begin{gathered} \text { Error, } \\ \% \end{gathered}$ |
|  |  |  |  |  | Motor input | Motor output (fan input) | Fan output |  |  |  |  |
| NASA-Ames Research Center 40- by 80-Foot | Single, closed/ rectangular | Closed/ flat oval | Conventional tunnel; multiple circular-arc turning vanes | 107.3 | Measured | Estimated from measured motor losses | Estimated <br> from <br> assumed <br> $n_{F}=97 \%$ | --- | 7.88 | 7.96 | 1.0 |
| NASA-Ames Research Center 7 - by 10-Foot | Single, closed/ rectangular | Closed/ rectangular | Some separation in primary diffuser; partial fan stall; multiple-circular-arc turning vanes; air exchanger available | 133.0 | Measured | Estimated <br> from <br> measured <br> motor losses | Estimated from assumed $n_{F}=85 \%$ | --- | 7.85 | 8.07 | 2.8 |
| Lockheed-Georgia LowSpeed (V/STOL Test Section) (ref. 20) | Single, closed/ rectangular | Closed/ rectangular | The larger of two tandem test sections was considered; test section vented | 52.3 | Measured | Estimated from assumed $n_{E}=95 \%$ | Estimated from assumed $n_{F}=95 \%$ | --- | 1.10 | 1.12 | 1.8 |
| Indian Institute of Science 14- by 9-Foot (at Bangalore) (refs. 21-23) | Non-return/ <br> flat oval | Closed / rectangular with corner fillets | Some dimensions for the estimate were scaled off of small drawings | 96.3 | --- | --- | --- | --- | $\begin{aligned} & 6.85 \\ & \text { (ref. 21) } \end{aligned}$ | 6.83 | -0.3 |
| Hawker Siddeley Aviation 15-Foot V/STOL (at Hatfleld) (ref. 24) | Non-return/ rectangular and circular | Closed/ rectangular | Basically a costeffective facility; some dimensions for the estimate were scaled off of small drawings | 45.7 | --- | --- | --- | Measured (ref. 24) | 2.38 | 3.97 | 66.8 |
| University of Washington 8 - by 12-Foot | Double, closed/ rectangular | Closed/ rectangular with corner fillets | Surprisingly high measured energy ratio | 117.7 | --- | --- | --- | Measured | 8.3 | 7.20 | -13.3 |
| NASA-Langley Research Center <br> 30- by 60-Foot (ref. 25) | Double, closed/ rectangular | Open/flat oval | High diffusion rates in some important components | 52.7 | Measured | Estimated from assumed $n_{E}=85 \%$ | Estimated from assumed $\eta_{F}=90 \%$ | -- | 3.71 | 4.73 | 27.4 |

a The quoted energy ratios are the best available and best achieved for each facility. The energy ratios of some facilities have dropped over the years
due to deterioration, leaks, soot build-up, etc.

(a) Several existing facilities.
Figure 1.- Diffuser design parameters.

(b) Recommended design region.

(a) Contraction geometry definition.
Figure 2.- Contraction design criteria.


(a) Turns with turning guide vanes.
Figure 3.- Local losses for turns as functions of turning angle.

(b) Turns without vanes.
Figure 3.- Concluded.

Figure 4.- Straight-walled diffuser expansion loss parameter variation with equivalent cone angle;

$$
K_{E X P}=K_{\text {Ref. }} 9-\frac{\lambda}{8 \sin \theta}\left(\frac{A R+1}{A R-1}\right)
$$




Main Program (PERFORM)

(a) Main program.

Figure 7.- Basic functional flow chart.

(a) Main program - Concluded.
Figure 7.- Continued.

(b) Data-checking subroutine.

Figure 7.- Continued.


Local Reynolds Number and Friction Coefficient Subroutine (FRICTN)

(c) Local speed and Reynolds number/friction coefficient subroutines.

Figure 7.- Continued.

(d) Section information output and plotting subroutines.

Figure 7.- Concluded.

(a) Complete source and data decks.
Figure 8.- Input deck setup.

(b) Sample data deck for three cases.

Figure 9.- Sample headings for input data sheets.

Figure 10.- Diagram of interior dimensions of NASA-Ames Research Center 40- by 80-Foot Wind Tunnel.

(a) Listing of input data cards.
Figure 11.- Test case information details.
 TUTAL CENTERLINE LENGTH $=588.74$ METERS

Figure 11.- Continued.
...CUNTINUED.... PAGE 3

(d) Section summary characteristics information.
Figure 11.- Continued.


(e) Summary information plots.
Figure 11.- Continued.



## Concluded.

Figure 11.- Continued.
5

NASA-AMES RESEARCH CENTER 40- BY 80-FOOT WIND TUNNEL

TUNNEL MASTER CONTROL OATA


| CASE TERMINATION CONDITIONS DATA |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
| AFTER 16 INPUT SECTIONS, AND ACCORDING TO THE FOLLOWING CONDITIONS -- |  |  |  |  |  |
|  | SUMMARY | PLOTTING AS | INPUT | VELDCITY- | RETURN |
|  | CHARACTERISTICS OUTPUT | A FUNCTION DF LEAGTH | $\begin{aligned} & \text { GATA } \\ & \text { TABULATION } \end{aligned}$ | OPTIMIZATION (FIXED POWER) | $\begin{gathered} \text { FOR NEXT } \\ \text { CASE } \end{gathered}$ |
| TERMINATICA-CCDE |  |  |  |  |  |
| DATA FIFLD IS CONTAINED IN |  |  |  |  |  |
|  | ---------- | +---------+ | +-------+ | +-----------+ | +------+ |
|  | YES | PRESS. LOSS, | YES (CHOSEN) | YES | NO |

(f) Annotated tabulation of input data.
Figure 11.- Continued.


Figure 11.- Continued.
NASA-AMES RES EARCH CENTER 40- BY 80-FOOT WIND TUNNEL
PAGE 7

(f) Annotated tabulation of input data - Concluded.
Figure 11.- Concluded.

$9.144 \times 10.06 \quad \begin{gathered}8.193 \times 9.107 \\ \text { rectangular }\end{gathered}$



(b) Lockheed-Georgia Low-Speed Wind Tunne1.
Figure 12.- Continued.

(c) Indian Institute of Science 14- by 9-Foot Wind Tunnel.
Figure 12.- Continued.
8.781

3) All lengths are along section centerlines.
(d) Hawker-Siddeley Aviation 15-Foot V/STOL Wind Tunnel at Hatfield.
Figure 12.- Continued.

(e) University of Washington 8- by 12-Foot Wind Tunnel.

> Figure 12.- Continued.

(f) NASA-Langley Research Center 30- by 60-Foot Wind Tunne1.
Figure 12.- Concluded.


[^0]:    *For sale by the National Technical Information Service, Springfield, Virginia 22161

[^1]:    ${ }^{1}$ Note that in this section, as throughout the report, all letter o's occurring in FøRTRAN names are shown with slashes, as $\emptyset$; all number zeros are shown without slashes.

[^2]:    PERFORFANCE SUMMARY -- $\quad$ ENERGY RATIO $=7.197$
    TOTAL NUMBER OF FANS
    $\begin{array}{ll} \\ \text { OUTPUT REQUIRED AVERAGE PER FAN } & \text { FAN EFFICIENCY }\end{array}$
    $\begin{array}{rr}\text { OUTPUT REQUIRED } & \text { AVERAGE PER FAN } \\ 1107045 . ~ W A T T S ~ & 553523 . ~ W A T T S\end{array}$

    TOTAL POWER -
    INPUT TC FLUW
    1051693. WATTS

[^3]:    
    
    
    
    
    

[^4]:    dOUBLE-RETURN, DPEN-TEST-SECTION WIND-TUNNEL PERFORMANCE

[^5]:    ${ }^{\text {a }}$ Default (X)" indicates the input is optional and defaults to $X$ if omitted. "Geom. Dep." indicates the input requirement is dependent on section geometry.
    "Optional" indicates the input may be selected and included as desired.
    "Required" indicates the input must be non-zero and included for all cases or the case will terminate due to input error.

[^6]:    a"Default(INT)" indicates optional input, dependent on section type, which defaults to an internallygenerated, geometry-dependent value if omitted.
    "Default(X)" indicates optional input, dependent on section type, which defaults to $X$ if omitted. "Geom. Dep." indicates the input requirement is dependent on section geometry.
    "Optional indicates the input may be selected and included as desired. Required indicates the input must be non-zero
    skipped and case terminated due to input error.
    "Sect. Dep." indicates the input requirement is dependent on section type.

