N86-14096

SPARGER SYSTEM FOR MMH-HELIUM VENTS

Based on a calculated vent flow rate and MMH concentration, a T1-59 program was run to determine total sparger nole area for a given sparger inlet pressure. Hole diameter is determined from a mass transfer analysis in the holding tank to achieve complete capture of MMH. In addition, based on oxidation kinetics and vapor pressure data, MMH atmospheric concentrations are determined 2 ft above the holding tank.

By: A. Rakow

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- A. MMH Vent Rates
- B. Sparger Program
- C. Mass Transfer Analysis in Tank
- D. Mass Transfer Analysis from Tank to Atmosphere
- E. Recommendations

A. MMH Vent Rates

A typical venting profile is given on the attached TPS. With 800 gallons of ullage and the assumption of ideal bahavior (in this pressure range helium exhibits heating rather than cooling upon expansion, but only about 0.06°C per atm; therefore, isothermal expansion is reasonable) the volumetric rate would be 1459 scfm during the first 10 seconds. With a vapor pressure of 0.957 psia the MMH concentration would be about 3400 ppm. At the completion of the vent the rate would be about 162 scfm and the concentration (assuming rapid vapor equilibrium during venting) 16,780 ppm.

For the purpose of designing a sparger, let's take $1500~\rm scfm$ as the max flow and $100~\rm psi$ as the sparger inlet pressure (either an analysis of Δ P in the vent system or pressure measurement during a vent is necessary to determine this parameter).

for the purpose of analysing dispersion of MMH to the atmosphere above the holding tank, we need to estimate the total amount of MMH vented. The average flow rate is 800 scfm for 3 minutes, i.e., we're venting 2400 scf which is 6.7 lb moles. At an average consentration of about 10,000 ppm MMH, we would be venting 0.067 lb moles of MMH.

8. Sparger Program

The attached sparger program will determine the total hole area and area distribution required for a liquid sparger.

Since our case is a gas dispersed into a liquid phase, we have to employ the gas orifice equation instead of equation (9) in the attached article by Mink.

The gas orifice equation:

$$W = 1891 \ Y \ d_0^2 \ C \left(\frac{\Delta P}{V_1}\right)^{\frac{1}{2}}$$

Where

W = lb/hr

$$\Delta P = psi$$

 $V_1 = ft^3/lb$
 $d_0 = orifice diameter, in.$

converts to

$$a = .26$$
 Q using a value of 0.6 for both Y $1830 \left[\Delta P/\rho\right]^{\frac{1}{2}}$

and C.

Therefore, equation (9) in the article by Mink probably overpredicts the area by a factor of 4. Keeping this in mind, the conditions of 1500 scfm and 100 psi were run, giving the following results:

For

= 11,220 GPM

B = 100 psi

= 20 psi (includes 9-ft head in tank)

D = 2 inches (Sparger I.D.)

A' = 0.011

B' = 0.011

C' = 7 ft

$$a = 0.72 in^2$$

Since this is an overprediction, we probably only need 0.18 in^2 total hole area (of course, it is extremely important to know the actual sparger inlet pressure; our value of 100 psi is used for illustrative purposes).

Mass Transfer Analysis in Holding Tank

Rate

The mass transfer rate to the tank can be approximated by:

$$N_{A} = \sqrt{\frac{4 P_{AB} V_{t}}{\pi d_{D}}} C_{A_{O}}$$

Where

 $N_A = \frac{q \text{ moles}}{CM^2 \text{ sec}}$ MMH transferred to water

 D_{AB} = diffusivity of MMH in H₂O

 V_{t}^{-} = terminal velocity of rise of drop $C_{A_{0}}$ = solubility of MMH in H₂O in moles/volume

= drop diameter

In order to use this equation we need to determine the drop diameter, do, and the terminal velocity.

The terminal velocity is given by:

$$V_{t} = \sqrt{\frac{2 \leq g_{c}}{d_{p} \ell_{z}} + \frac{g d_{p}}{2}}$$

Where $\ensuremath{\mbox{\ensuremath{\mbox{Δ}}}$ is the surface tension of water and $\ensuremath{\mbox{$\rho_{\sigma}$}}$ is liquid density.

The particle (drop) diameter in inches is given by:

$$d_p = 0.279 (Re_0)^{-0.05}$$

based on a jet correlation (p. 141, Treybal).

Using an upper limit of 50,000 on Reo, we get 0.16 inches for the bubble diameter. This gives a value of 0.77 ft/sec for Vt.

Plugging these values into the equation for NA gives a very high rate of mass transfer indicating that most, if not all of the MMH, should be absorbed.

2. Orifice Diameter and Number of Holes

The orifice Reynolds's number
$$Re_0 = \frac{d_0 V_0}{\mu}$$

Where V_0 is the velocity in the orifice and $\,\rho$ and μ are density and $\,^{\bullet}$ viscosity, respectively.

Using the criterion of $Re_0 = 50,000$, we get the following two equations:

For 1500 scfm

1.
$$n d_0 = 11.36$$

2.
$$\pi \left(\frac{d}{2}\right)^2 n = a = 0.18 \text{ in}^2$$

where n = no. of holes

Therefore
$$\frac{\pi d_0}{4}$$
 (11.36) = 0.18

or
$$d_0 = 0.020$$
 inches

$$\cdot \cdot n = 568$$

So $\underline{568}$ holes at $\underline{0.02}$ inches per hole ($\underline{1/50}$ ") should do the job in this case.

D. Mass Transfer Analysis from Tank to Atmosphere

This analysis is based on the following assumptions:

- 1. 0.067 lb moles of MMH are completely absorbed from the sparger during a vent in a well mixed tank of water 10' X 5' X 9' filled to the 6-foot level.
- 2. MMH is depleted by two mechanisms
 - a. diffusion from the gas liquid interface through 3 ft of air to the top o` the tank where it is swept away (zero concentration at top)
 - b. in the tank MMH disappears by oxidation at a rate of 100 ppm/day (This number is an average of open pan tests done by Eric Miller and data in the Florida Tech handbook.)

- 3. the Henry's law relationship for MMH is y=mx, where m=0.05, and y and x are the mole fractions of MMH in air and H_2O , respectively
- 4. equilibrium exists at the gas liquid interface
- 5. diffusion in the gas phase is quasi-steady and unidirectional

First, let's compute the initial mole fraction of MMH in the tank, \mathbf{x}_{I_0} , in ppm:

300 ft³ H₂0 X 62.4
$$\frac{1b}{ft^3}$$
 X 1 $\frac{1b \text{ mole}}{18 \text{ lb}}$ = 1040 lb moles H₂0 $\frac{.067}{1040}$ = 64 ppm = X ℓ ₀

The diffusion rate in the gas phase is given by

$$N_{A} = \frac{\int_{MMH-Air} A C (0.05)(X_{I})}{I}$$

Where NA - flux in moles/area time

MMH-Air - diffusivity of MMH in air

A - cross sectional area of tank

C - gas molar density

X - mole fxn of MMH in tank at any time, t

L - distance between the gas-liquid interface and t

 $\hat{\mathcal{L}}$ - distance between the gas-liquid interface and the top of the tank

Using a value of 0.2 cm²/sec = 0.775 $\underline{\text{ft2}}$

for $\mathcal{L}_{\text{MMH-Air}}$ we get

$$N_A = 0.00177 \times_{g} (t) \frac{\text{moles}}{\text{hr}}$$

Now we can write a mass balance for MMH in the tank to compute $X_{\mbox{\it g}}$ as a function of time, t.

rate of accumulation = - rate of depletion - rate at which MMH leaves of MMH in tank by oxidation thru interface

OT

$$\frac{1040 \text{ dX}_{\ell} = -10^{-4} (1040)}{\text{dt}} \cdot \frac{1 \text{ day}}{\text{day}} - 0.00177X_{\ell}$$

where t is in hours

The solution to this equation is:

$$X_{\ell} = 2.35(e^{-1.7} \times 10^{-6}t_{-1}) + X_{0} e^{-1.7} \times 10^{-6}t$$

or
$$X_{\ell} = (X_{\ell_0} + 2.35) (e^{-1.7} \times 10^{-6}t) - 2.35$$

for complete depletion $X_{\ell} = 0$ and $X_{\ell_0} = 64 \times 10^{-6}$

so
$$e^{-1.7 \times 10^{-6}t} = \frac{2.35}{2.35 + 64 \times 10^{-6}} = 0.99997$$

$$-1.7 \times 10^{-6}t = -0.0000272337$$

Lastly, let's look at the concentration two feet above the gas-liquid interface initially (this will drop off to zero at 16 hours),

The concentration profile above the liquid is:

$$-\frac{dy}{dx} = 1.06 \times 10^{-6}$$

initially at the gas liquid interface we have $Y_A = 0.05 (64 \times 10^{-6})$ (where X = 0)

$$Y_A = 3.2 \times 10^{-6} = 3.2 \text{ ppm}$$

so
$$\frac{3.2 - Y_2}{2ft} = 1.06$$

$$Y_{2ft} = 1.08 ppm$$

Any effort to speed up the oxidation process such as low ph should enable the maintenance of low concentration above the tank. Forced convection would also do the job.

E. Recommendations

- 1. Determine the holding tank inlet pressure during a vent
- Measure MMH in the tank and in the vapor space above the tank during a vent with current system
- Do 1 and 2 after sparger is installed
- Consider agitation if bubble coalescence occurs
- 5. Determine treatment schedule and/or second stage process to meet EPA standards for atmospheric MMH

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Hole-area distribution for liquid spargers

This TI-59 program does the tedious calculations involved in designing spargers, and prints (or displays) the results.

William H. Mink, Battelle Columbus Laboratories "

☐ In many operations, it is necessary to sparge a liquid or gas into another fluid; frequently, a uniform distribution is desired. Since, owing to frictional losses, the pressure decreases as the fluid flows down the sparger pipe, the hole area must increase along the length of the sparger to maintain uniform flow from it.

H.W. Coopert has developed a series of relationships that are useful in calculating sparger hole area for uniform distribution. It has been pointed out that these equations are in error because the originator neglected to include the effect of velocity head. This velocity-head effect can be significant. Only when the pipe L/D ratio is high or when flowrates are low do the Cooper equations provide an approximation. The author has developed a new program, breed on a stepwise calculation, which yields correct hole sizes regardless of L/D ratio or flowrates.

Calculational approach

The pressure causing flow through any orifice is the difference between the static pressure at the point of the orifice and the ambient pressure. The static pressure is the total pressure less the velocity head and any frictional loss:

$$P_t = P_t - P_{\phi h} - P_f \tag{1}$$

For the sparger, the total inlet pressure is:

$$P_{i,in} = P_{a,in} + P_{\phi h,in} \tag{2}$$

In Eq. (1) and (2),

$$P_{\rm pk} = \frac{V^2 \rho}{2g(144)} = \frac{V^2 \rho}{9,274} \tag{3}$$

$$V = \frac{0.4085Q}{d^2} \tag{4}$$

 $P_t = \frac{FV^2\rho L/10}{d(193)} \tag{5}$

"For information about the author see p. 225

"Couper, Herbert W., Area Allocation for Distributor Pipm, Chrs. Eng., Oc

The friction factor, F_n depends on the Reynolds number, N_{Re} :

$$F = 16/N_{Re}$$
, if $N_{Re} < 2{,}100$ (6)

$$F = 0.0035 + 0.264(N_{Re})^{-0.42}$$
, if $N_{Re} \ge 2,100$ (7)

where

$$N_{Re} = \frac{124 \, rep}{\mu} \tag{8}$$

The orifice area can be calculated by the orifice equation, which can be simplified to:

$$a = Q/1830[(P_a + P_a)/\rho]^{1/2}$$
 (9)

A program has been developed for the TI-59/PC-100A, using the above procedure. The program calculates the hole area for each 10% of sparger length (using Eq. (9)).

Eq. (9)).

The program is shown in Table I. Certain constants for the calculations and for the alphanumeric operations must be stored in the data registers. These proshown in Table II. Once the calculator has been programed and the constants stored, both the program and data register contents may be recorded on magnetic cards for future use.

User instructions for calculation of the sparger hole areas that will achieve uniform flow distribution are given in Table III.

When running this program, the printer first prints out the entered data. Next, it prints the hole area for each 10% of sparger length, and finally it prints the total parger hole area.

Procedure

For the calculation of the hole area in the first section (first 10% of longth), the flowrate down the pipe is set at the extend union. For each subsequent hole-area calculations

Table I

| ### Code Exp | Control Code Ray 060 69 DP 061 05 05 062 98 ABV 063 98 ABV 064 91 21 065 69 DP 067 04 RCL 065 69 DP 067 04 RCL 066 69 DP 067 04 RCL 067 04 RCL 073 22 074 69 DP 075 04 RCL 077 078 69 DP 078 080 23 DP 079 083 RCL 077 078 09 DP 081 23 DP 082 69 DP 083 084 03 RCL 085 69 DP 085 69 DP 087 087 25 DP 087 089 24 DP 098 04 RCL 098 04 RCL 099 05 06 RCL 099 05 06 RCL 091 04 RCL 092 05 06 RCL 093 05 06 RCL 094 06 DP 095 06 RCL 095 07 7 7 098 09 DP 100 05 06 RCL 101 06 06 DP 102 07 7 7 118 09 DP 111 02 2 113 07 DP 115 04 RCL 116 07 DP 117 07 DP 118 06 DP | Code Rev 120 69 0P 000 121 00 00 122 98 124 19 0P 125 19 0P 126 69 0P 127 00 0P 128 19 0P 128 19 0P 128 19 0P 128 19 0P 129 00 04 129 00 04 129 00 05 130 09 09 131 00 06 132 00 09 133 09 09 134 09 09 135 00 00 00 00 00 00 00 00 00 00 00 00 00 | 180 04 4 181 95 = 182 42 STD 183 14 14 184 02 2 185 01 1 186 00 0 187 00 0 188 32 XIT 189 43 RCL 190 13 13 191 65 x 192 01 1 193 02 2 194 04 4 195 65 x 196 43 RCL 197 04 04 198 65 x 196 43 RCL 200 06 06 201 55 + 202 43 RCL 200 06 06 201 55 + 202 43 RCL 200 06 06 201 55 + 202 43 RCL 203 05 05 15 15 15 204 85 + 205 01 1 206 6 201 55 + 202 43 RCL 203 05 05 204 85 + 205 01 1 206 6 217 05 16 218 15 15 216 95 = 217 42 STD 221 06 6 221 3 55 + 222 43 RCL 223 16 16 224 65 X 223 16 16 223 17 12 12 12 12 12 12 12 12 12 12 12 12 12 | 240 03 3 3 241 00 0 0 242 95 = 0 242 95 = 0 244 17 17 17 245 87 1FF 246 01 01 247 89 4 248 77 CE 250 29 CP 251 43 RCL 253 75 RCL 253 75 RCL 255 43 RCL 255 43 RCL 255 43 RCL 264 03 03 265 95 = 0 264 03 43 RCL 264 03 03 265 95 = 0 267 77 258 81 | 300 73 RC+ 301 09 09 302 69 DP 303 04 04 304 43 RCL 305 10 10 306 44 SUM 307 08 08 308 58 F1X 309 03 03 310 69 DP 311 06 06 312 58 F1X 313 09 09 314 01 1 315 44 SUM 316 09 09 317 61 GTD 318 69 DP 319 76 LBL 320 68 NDP 321 98 RDV 321 98 RDV 322 43 RCL 323 30 30 324 69 DP 325 01 01 326 43 RCL 327 31 31 328 69 DP 329 02 02 330 43 RCL 327 31 31 328 69 DP 325 01 01 326 43 RCL 327 31 31 328 69 DP 329 02 02 330 43 RCL 327 31 31 31 31 31 328 69 DP 329 02 02 330 43 RCL 327 31 31 31 31 31 328 69 DP 329 02 02 330 69 DP 320 69 DP 320 69 DP |
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| 1 | Enser program (use standard partitioning) | | | |
| 2 | Enter data | Flow to spergar, gom | A | GPM |
| | | Pressure to sparger, psia | | PAIN |
| | | Ambient pressure, psia | C | PAAM |
| | | Sperger I.D., in, | D | 1D |
| | | Sperger fluid viscosity, cP | A' | VISC |
| | | Sperger fluid density, Ib/It ³ | | DEN |
| | | Sporger length, ft | C | L |
| 3 | Calculate | | E | Hole area per 10% of sparger length, and total hole area, in,2 |
| | | | R/S* | Hale area per 10% of sparger length, and total hale area, in,2 |

lation, the flowrate down the pipe is reduced by 10% of the entered value.

The velocity is calculated by Eq. (4) (Step 157 in the program shown in Table I), the velocity head by Eq. (3) (Step 172), and the Reynolds number by Eq. (8) (Step 189). The appropriate equation for friction factor is selected [Eq. (6) or (7)]. The frictional loss is calculated by Eq. (5) (Step 222).

The pressure difference, $P_a - P_{et}$ is calculated using Eq. (1) and the entered ambient pressure. This difference is then used to calculate the area, using Eq. (9). This series of calculations starts at Step 251.

It should be noted that, since the entered P_{in} is a static pressure, $P_{i,in}$ must be calculated using Eq. (2). (Since holes are assumed to be located at the center of each section, the frictional loss for the first section takes place over only 5% of the pipe length rather than '0% as for the subsequent sections. To adjust for the program, which subtracts the frictional loss for 10% of the pipe length for each hole calculation, the initial total pressure used in the calculation is increased by one-half of the initial frictional loss.)

Sample problem

Cooper demonstrated the calculation of hole area with the following example:

Consider 2,000 gal/min of water at 25 psia and 95°F, which is to be sprayed onto a distributor plate via holes in an internal pipe that is 10 ft long. Determine the area allocation for a 6-in. pipe. Tower pressure is 24.5 psia In this case, $\rho = 62.4$ lb/cu ft, $\mu = 6.76$ cP.

Table IV shows the printout for this example. The areas calculated differ substantially from those calculated by Cooper; in this example, velocity head is an important factor.

"The Rynalds number is increased by 1 in the program so that Eq. (3) does not give an inflante require (easing the calculators in indicate an error condition as the end of the calculation) when 1' in B. The addition of 1 has negligible effect on the calculation results.

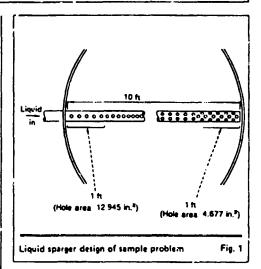


Fig. 1 shows, diagramatically, the application of the calculated results. (The hole size and spacing shown are for illustrative purposes and do not relate to the sample problem.) In this example, the hole area could be obtained by several combinations of hole sizes and numbers of holes. One possible arrangement would be to have 16 1-in, holes over the first 10% of length, 11 1-in holes over the next 10% of length, and finally 6 1-in holes over the last 10% of length.

over the last 10% of length.

If the TI-59 is to be used without the printer, it will be necessar; to make one change in the program, as follows:

1. Enter the program as shown in Table I

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HOLE-AREA DISTRIBUTION FOR LIQUID SPARGERS

| Sample calculation | | Table IV |
|---|--|----------|
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| 2000. 25. 24. 5 6. 0648 0. 75 32. 4 10. | GFH P_IH P_AH ID VISC DEH L | |
| IN2/10%L 12, 945 8, 697 7, 101 6, 230 5, 680 5, 307 5, 047 4, 866 4, 746 | % 0/10 /20 /30 /40 /50 /60 /70 /80 | <i>:</i> |
| 4.677 TOTAL HOLE ARE 65.296 | 100 4, IH² | |

2. Make a program addition by pressing the following key sequence: GTO; 344; LRN; 2nd Del; 2nd Del; LRN; GTO; 312; LRN; 2nd Inr; R/S; LRN.

With this change, the calculator will display the hole area for the first 10% of sparger length after E is pressed. Repeatedly pressing R/S will cause the calculator to display the hole area for each of the next sparger segments in turn and, finally, the total hole area. As an aid to the operator, for identification purposes, the hole areas for the sparger segments are displayed in Fix 3

Nomenclature

- Orific: area, in.2
- Sparger pipe I.D., in.
- D Sparger pipe I.D., ft
- Pipe length, ft
- Ambient pressure (pressure sparger is discharging into), psia
- Initial (entered) static pressure, psia
- Frictional loss, psi
- Static pressure, peia
- Total pressure, psia
- Velocity head, psi
- Flowrate entering sparger pipe, gpm
- Velocity in sparger pipe, ft/s Fluid density, lb/ft³
- Fluid viscosity, cP

mode while the total hole area is displayed in Fix 9 mode

Should the data entered result in an impossible solution (such as might occur if the sparger diameter were too small to permit the required flow at the specified pressures), the calculator will print out "SELECT LARGER PAIN OR ID."

For HP-67/97 users

The HP version of the program closely follows the TI version. Table V prvides a listing of the HP program, user instructions for the HP version are identical to those provided in Table III, with A', B', and C', replaced by a, b, and c, respectively.

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| 12 | | | i23° | 7 | 67 | 144 | STOE | 25 15 | 165 | ۲. | - 35 | 186 | • | 6 |
| 63 | Kilm | 3€ 17 | 124 | A | -35 | 145 | • | -53 | 100 | KCLE | 36 12 | 187 | é | ð |
| 64 | 1 | 61 | 125 | 4 | *-24 | 146 | 1 | 4: | 167 | 3 | 0 3 | 188 | • | -3 |
| P5 | * | 8.5 | 126 | (H\$ | -22 | 147 | ENTI | -2: | 144 | • | -35 | 189 | \$107 | 35 € |
| li. | 4 | - 35 | 127 | \$100 | 35 14 | 146 | 3 | C3 | :69 | • | -24 | 150 | KIK | 2 |
| | 8515 | 36 65 | 128 | SCLD | 36 14 | 146 | + | -24 | 156 | • | -45 | 191 | SLEL? | 21 6 |
| 08 | 1,8 | 53 | 129 | 2 | # 2 | !5é | ľ | 31 | 171 | 11 | 54 | 192 | RCLZ | ₩ 6 |
| ė.s | • | -35 | 138 | + | -74 | :51 | Riti | 36 14 | 172 | - 14 | . 54 | 193 | RCL4 | 3i 6 |
| ı e | • | -55 | 131 | CHZ | -22 | 152 | 2 | ė.: | 173 | ECL3 | 36 83 | 194 | | -3 |
| ; | aci e | še 1ž | 132 | RCLD | 36 14 | :53 | + | -24 | 174 | 7 | -35 | 195 | · | - 6 |
| 12 | RCL9 | 36 65 | 133 | Χs | 53 | 154 | CHS | -22 | 175 | FRIX | -14 . | 196 | • | • |
| : ; | χı | 53 | 134 | 4 | 64 | 155 | RELE | 36 15 | 176 | SFC. | i6-11 | 197 | + | -2 |
| | 7 | -35 | 135 | • | -24 | 156 | - | -45 | 177 | G e | 16 22 🙀 | 198 | STOG | 35 € |
| 4 | ž | 0 č | 136 | RCLC | 36 15 | 157 | 1 | 6; | 176 | CF1 | 16 22 0 1 | 199 | RTh | 2 |
| 14 | | | 137 | 3 | 0.5 | 156 | EHII | -2: | 175 | R-S | 51 | 246 | E-5 | 5 |
| 14 15 | ; | 67 | | | | | | 4- | 180 | SLEL1 | 2: 01 | ı | | |
| 14 15 16 | | 67 -35 | 138 | y. | 31 | 15.9 | 3 | 63 | 11.00 | | | ŀ | | |
| 14 15 16 17 | 7 | | | | 62 | 15.9 166 | j | -24 | 181 | RCLI | 3€ 61 | | | |
| 14 15 16 17 16 | , | -35 | 138 | y= | | | | -24 31 | 181 182 | RCL I | 3€ 6 1 -62 | | | |
| 14 15 16 17 16 19 | , , | -35 -51 | 138 | y= 2 | 62 | 164 | • | -24 | 181 | RCL I | 3€ 61 | | | |

ADDENDUM

SPARGER SYSTEM FOR MMH-HELIUM VENTS

On page 4 the Sparger program for an example vent condition was run giving a result of a = 0.72 in 2 (total Sparger area). It was decided to run the same program for the same conditions, except for the Sparger inlet pressure which was 100 psi previously and is now taken to be 21 psi (instead of a Δ P of 80 psi as in the previous case we now have a Δ P of 1 psi through the Sparger).

The results are the following:

| Hole Area, in ² /10% of Sparger length, L | <u>%L</u> |
|--|-----------|
| . 664 | 10 |
| .605 | 20 |
| .562 | 30 |
| .529 | 40 |
| . 503 | 50 |
| .484 | 60 |
| . 468 | 70 |
| . 457 | 80 |
| . 449 | 90 |
| . 444 | 100 |

Total Area = $5.166 \text{ in}^2 = a$

Again, since the program is for liquids we need to multiply these numbers by 1/4 to get the areas needed for a gas vent. Also, in this case the area required continuously decreases as we go down the Sparger to compensate for velocity head loss.

Using this result, if we now refer to page 5 we have the following two equations for the first 10% of Sparger:

2.
$$\pi \left(\frac{\text{do}}{2}\right)^2 n = a = \frac{.664}{4} = .166 \text{ in}^2$$

Therefore
$$\frac{\pi \, do}{4} \, (1.136) = .166$$

and do = .184 inches
$$n = 6.17 \text{ holes}$$

Table I shows the results for 10% increments in length down the Sparger.

TABLE I. HOLE SIZE AND NUMBER REQUIRED

| Percent | n (no. of holes) | do, in |
|---------|------------------|--------|
| 10 | 6.17 | .184 |
| 20 | 6.80 | .167 |
| 30 | 7.29 | .156 |
| 40 | 7.75 | .147 |
| 50 | 8.15 | .139 |
| 60 | 8.47 | .134 |
| 70 | 8.76 | .130 |
| 80 | 8.97 | .127 |
| 90 | 9.13 | .124 |
| 100 | 9.23 | .123 |

Note, that we need far fewer holes of a larger diameter than in the previous case.