

**Development of an Algebraic Stress/Two-Layer Model  
for Calculating Thrust Chamber Flow Fields**

By

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**ABSTRACT**

Following the consensus of a workshop in Turbulence Modeling for Liquid Rocket Thrust Chambers, the current effort was undertaken to study the effects of second-order closure on the predictions of thermochemical flow fields. To reduce the instability and computational intensity of the full second-order Reynolds Stress Model, an Algebraic Stress Model (ASM) coupled with a two-layer near wall treatment was developed. Various test problems, including the compressible boundary layer with adiabatic and cooled walls, recirculating flows, swirling flows and the entire SSME nozzle flow were studied to assess the performance of the current model. Detailed calculations for the SSME exit wall flow around the nozzle manifold were executed. As to the overall flow predictions, the ASM removes another assumption for appropriate comparison with experimental data, to account for the non-isotropic turbulence effects.

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11th Workshop for CFD Applications in Rocket Propulsion

April 20-22, 1993

NASA- Marshall Space Flight center

- **Improve Predictive Capabilities of Turbulent Transport in Thrust Chamber**
- **Non-Isotropic and Compressibility Effects are the Focus of the Study**
- **Simplified Reynolds Stress Modeling**
- **Further Modeling in Turbulent Transport of Thermal Energy and Chemical Species -  $\overline{u_i C_i}$  and  $\overline{u_i T_i}$  etc.**

## **Motivation and Objective**

- Higher Order Models Are Desirable For Calculating Thrust Chamber

### **Flow Fields**

- 1991 Thrust Chamber Turbulence Modeling Workshop

- To Develop a Simplified 2nd-Order Turbulence Model For Thrust Chamber

### **Flow Calculation**

- Near wall treatment
- Efficiency and stability

## APPROACH

- PDE's for Reynolds stress  $\overline{u_i u_j}$  can be derived .  
Modeling any unknown in terms of Reynolds stress, the mean strain rate etc.
- Simplifications of the Differential Reynolds stresses Equations  
— Algebraic Stress Model(ASM)
- Non-linear constitutive relations (Spezial)

## APPROACH (DRS Equation)

- Differential Reynolds Stress Equation

$$\frac{D}{Dt} \overline{\rho u_i u_j} = P_{ij} + D_{ij} + \pi_{ij} + C_{ij} - \epsilon_{ij}$$

$$P_{ij} = -\overline{\rho} \left[ \overline{u_i u_k} \frac{\partial \overline{u_j}}{\partial x_k} + \overline{u_j u_k} \frac{\partial \overline{u_i}}{\partial x_k} \right]$$

production

$$D_{ij} = \frac{\partial}{\partial x_k} \left[ \overline{\rho} \overline{u_i u_j u_k} + \delta_{ik} \overline{u_j p} + \delta_{jk} \overline{u_i p} - (\mu \overline{S_{ik} u_j} + \mu \overline{S_{jk} u_i}) \right]$$

diffusion

$$\pi_{ij} = p' \left( \frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right)$$

pressure-strain correlation

$$C_{ij} = - \left[ \overline{u_i} \frac{\partial \overline{p}}{\partial x_j} + u_j \frac{\partial \overline{p}}{\partial x_i} \right]$$

compressibility

$$\epsilon_{ij} = \mu \left[ \overline{S_{ik} \frac{\partial u_j}{\partial x_k}} + \overline{S_{jk} \frac{\partial u_i}{\partial x_k}} \right]$$

dissipation

## APPROACH --- ASM

- Similitude Principle ( Mellor and Yamada )

$$P_{ij} - \frac{2}{3} \delta_{ij} P_k + \phi_{ij} + C_{ij} \cong 0$$

- Algebraic Reynolds Stress Model of Rodi

$$\frac{D}{Dt} \overline{\rho u_i u_j} - D_{ij} \cong \frac{\overline{u_i u_j}}{k} \left[ \frac{Dk}{Dt} - D_k \right]$$

$$\Rightarrow \left[ P_{ij} + \phi_{ij} + C_{ij} - \epsilon_{ij} \right] = \frac{\overline{u_i u_j}}{k} \left[ P_k - \epsilon \right]$$

## APPROACH ---Pressure-Strain Term

$$\tau_{ij} = \overline{p \left( \frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right)}$$

return to isotropy	$= -C_1 \frac{\varepsilon}{k} (\overline{\rho u_i u_j} - \frac{2}{3} \delta_{ij} k)$	Rotta Model
rapid term	$-C_2 (P_{ij} - \frac{2}{3} \delta_{ij} P_k)$	IP Model
wall damping term	$+ \tau_{ijw}$	Lumped with the two-layer model
in which	$P_k = \frac{1}{2} P_{ii}$	



$$\bullet \quad \frac{\overline{\rho u_i u_j}}{k} = \frac{(1-C_2)(P_{ij} - \frac{2}{3}\delta_{ij}P_k) - \frac{2}{3}\delta_{ij}C_k + C_{ij}}{P_k + \epsilon(C_1 - 1) + C_k} + \frac{2}{3}\delta_{ij}$$

where

$$P_{ij} = -\bar{\rho} \left[ \overline{u_i u_k} \frac{\partial \bar{u}_j}{\partial x_k} + \overline{u_j u_k} \frac{\partial \bar{u}_i}{\partial x_k} \right]$$

$$P_k = \frac{1}{2} P_{ii}$$

## k- ε Equations

- $$\frac{\partial \rho k}{\partial t} + \frac{\partial}{\partial x_j} (\rho U_j k) = \frac{\partial}{\partial x_j} \left( C_{kp} \frac{k}{\varepsilon} \overline{u_j u_l} \frac{\partial k}{\partial x_l} \right) + \mu_t G - \rho \varepsilon$$
- $$\frac{\partial \rho \varepsilon}{\partial t} + \frac{\partial}{\partial x_j} (\rho U_j \varepsilon) = \frac{\partial}{\partial x_j} \left( C_{\varepsilon p} \frac{k}{\varepsilon} \overline{u_j u_l} \frac{\partial \varepsilon}{\partial x_l} \right) + \frac{\varepsilon}{k} (C_{\varepsilon 1} \mu_t G - C_{\varepsilon 2} \rho \varepsilon)$$

where the production term  $\mu_t G$  takes the form

$$\mu_t G = -\rho \overline{u_i u_j} \frac{\partial U_i}{\partial x_j} - \frac{\mu_t}{\rho^2} \frac{\partial \rho}{\partial x_j} \frac{\partial P}{\partial x_j}$$

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**Turbulence model constants**

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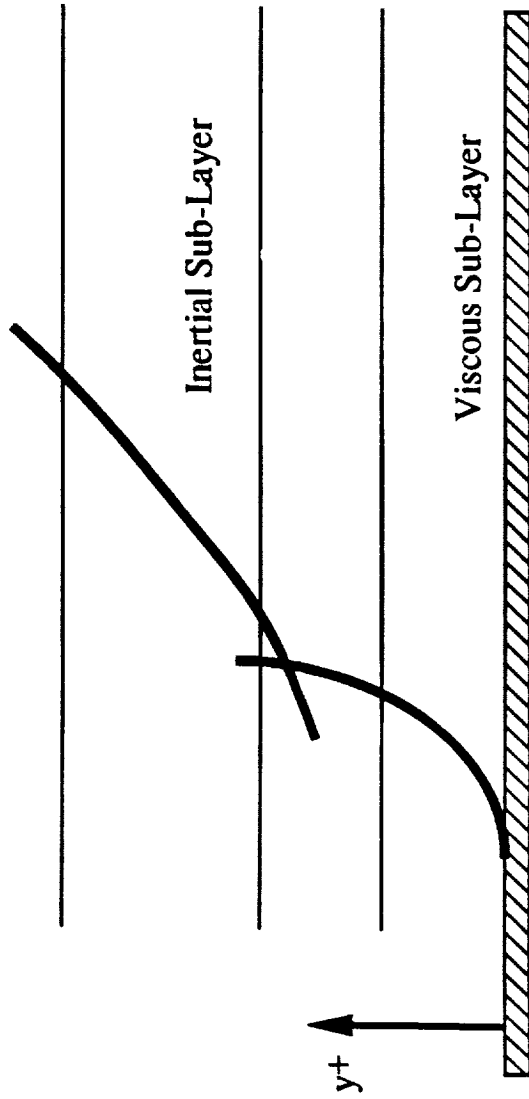
$C_{\epsilon 1}$     $C_{\epsilon 2}$     $C_k$     $C_\epsilon$     $C_1$     $C_2$

---

**1.45   1.92   0.22   0.15   2.5   0.5**

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# Two-Layer Wall Treatment



Outer Layer ---  
resolved by  
ASM

Inner Layer ---  
Patched with a One-Equation  
Model

Matching at

$$R_k = \frac{k^{1/2}y}{\mu} = 200$$

# METHODOLOGIES

$$v_t = u' l' = C_\mu \kappa^{1/2} l_\mu = C_\mu \frac{\kappa^2}{\varepsilon} = C_\mu \frac{l_\varepsilon}{\kappa^{3/2}} \kappa^2$$

within Inertia Sublayer  $\varepsilon = C_\mu^{3/4} \frac{\kappa^{3/2}}{l_\mu} = \frac{\kappa^{3/2}}{l_\varepsilon}$

$$l_\mu = C_l y \left[ 1 - \exp \left( - \frac{R_\kappa}{A_\mu} \right) \right] \rightarrow \text{to be used in Eddy Viscosity}$$

$$l_\varepsilon = C_l y \left[ 1 - \exp \left( - \frac{R_\kappa}{A_\varepsilon} \right) \right] \rightarrow \text{to be used in } k \text{-equation}$$

$$C_l = \kappa C_\mu^{-3/4}$$

Matching at  $R_\kappa = \frac{\kappa^{1/2} y}{\nu}$

## **IMPLEMENTATIONS**

- **Implemented into MAST-2D**
- **Non-Staggered Grids, Sequential Solver**
- **Chakravarthy-Osher TVD Scheme**
- **PISO-C Algorithm**
- **Conjugate Gradient Matrix Solver**
- **Time Marching**

## **Validations**

- **Incompressible & Compressible Flat Plate  
Cooled & Heated Wall, Up To Mach 10**
- **Incompressible & Compressible Recirculating Flows**
- **Incompressible Swirling Flows**
- **Thrust Chamber Flows**

# Compressible Flat Plate Flow

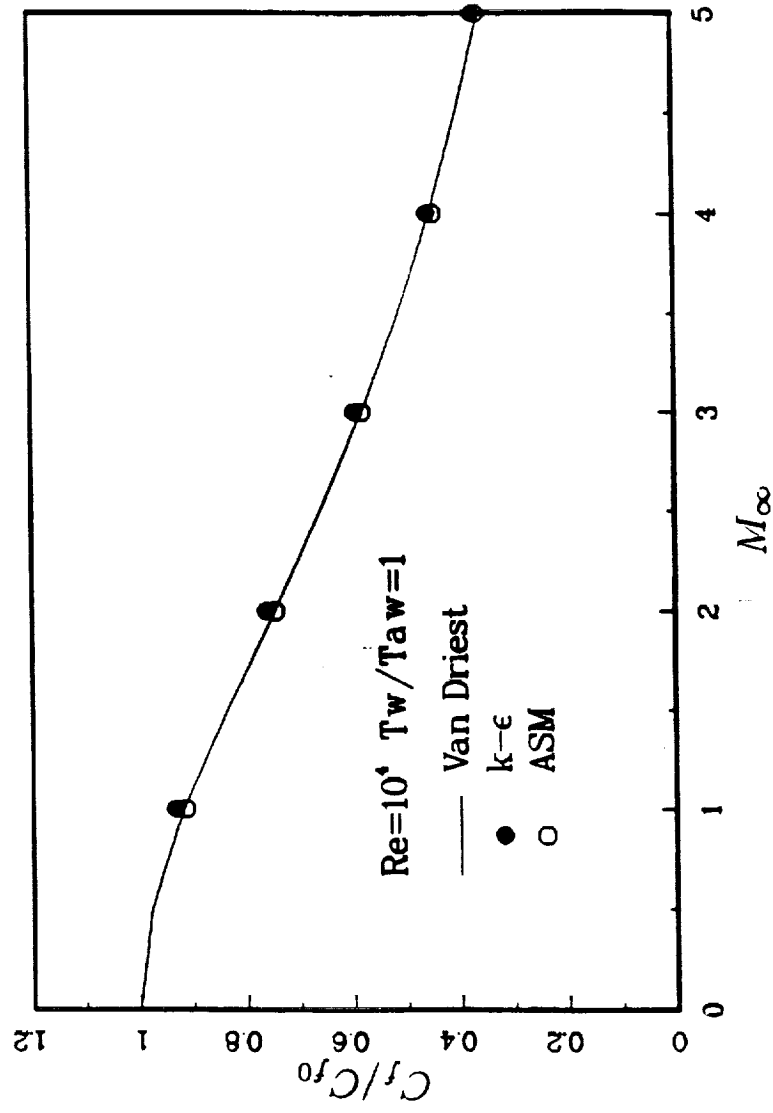


Fig. 1 Variation of  $C_f/C_{f0}$  with  $M_\infty$  for adiabatic wall boundary condition.



# Compressible Flat Plate Flow

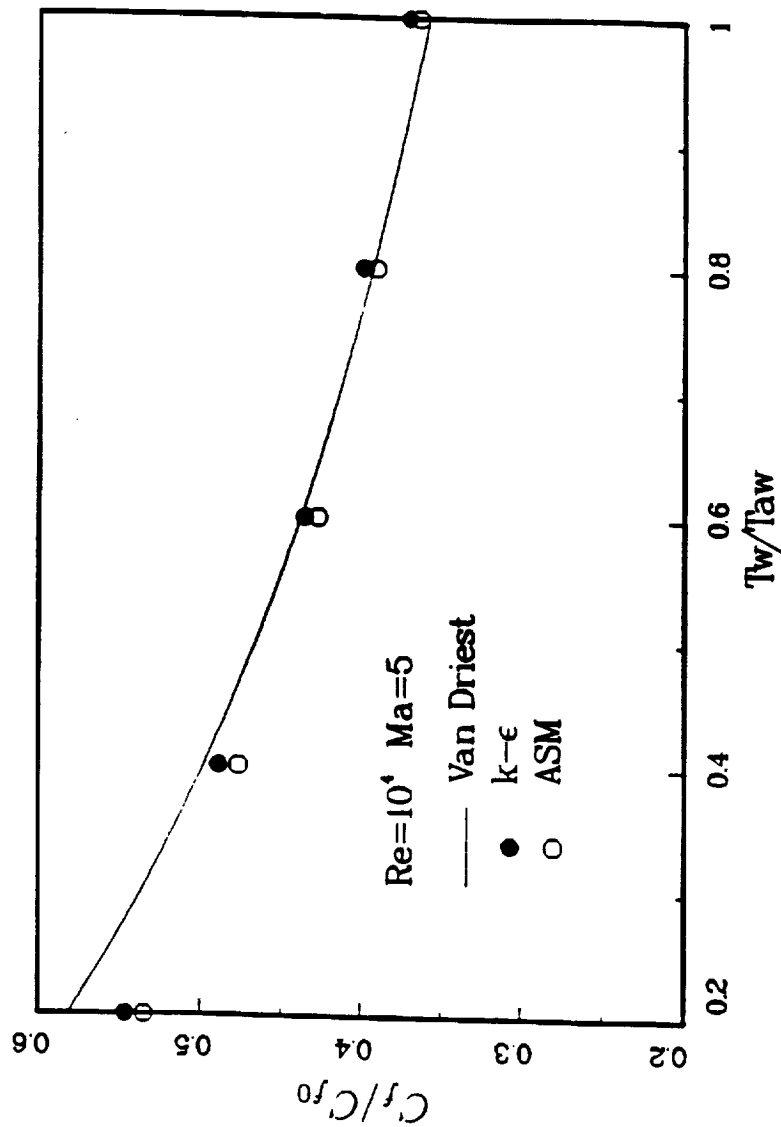


Fig. 2 Variation of  $C_f/C_{f0}$  with  $T_w/T_{aw}$  for  $Ma_\infty = 5.0$ .

# Compressible Flat Plate Flow

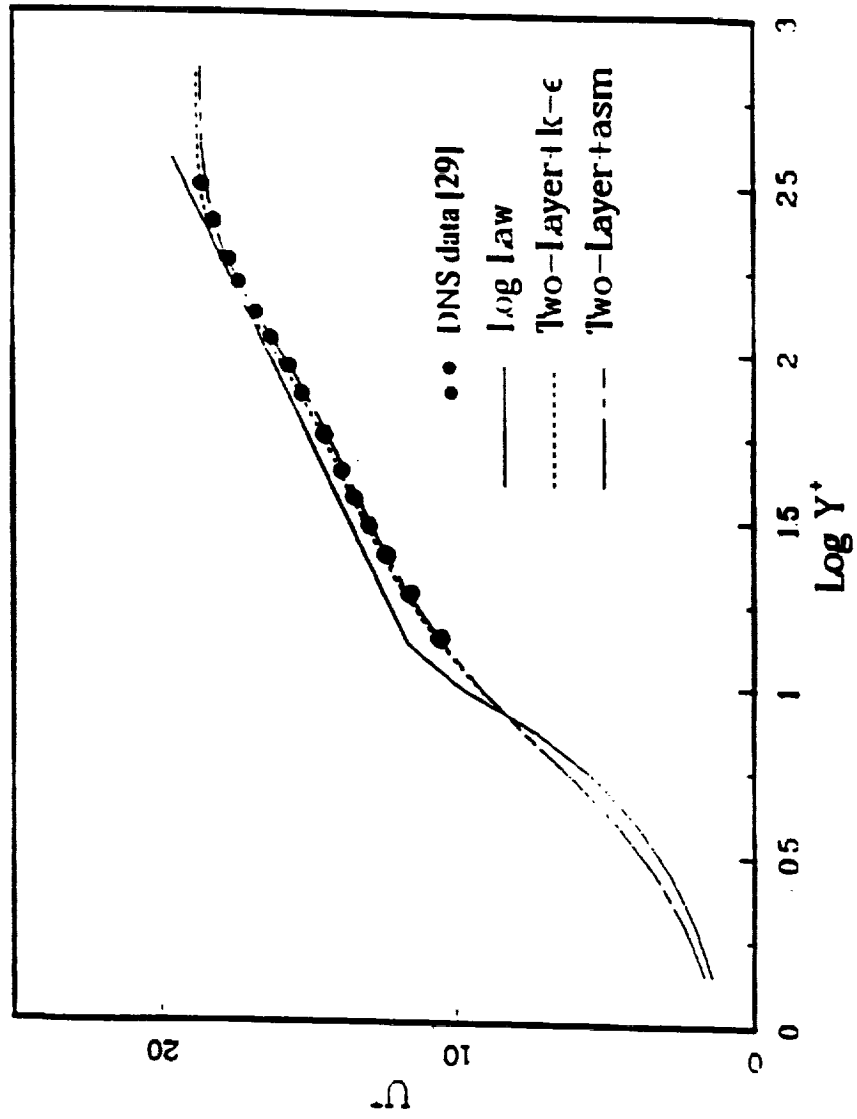
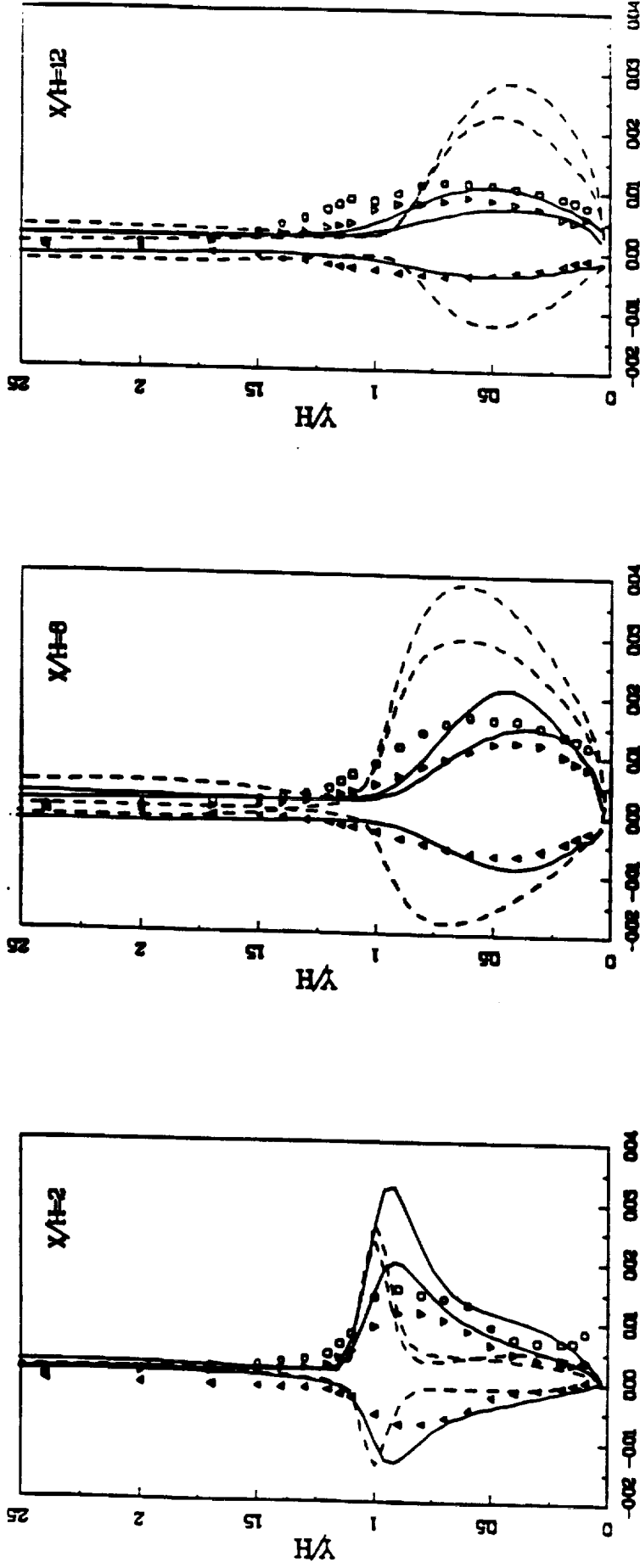


Fig. 3 Semi-log plots of  $u_c^+$  for adiabatic wall boundary condition.

# Backward Facing Step



Attachment length:  $k-\epsilon$  4.67H  
 ASM 5.94H  
 EXP.  $\sim 6H$

Fig. 5 Reynolds stress profiles for the backward-facing step turbulent flow (9:1), with data from [31]

# Confined Swirling Flows

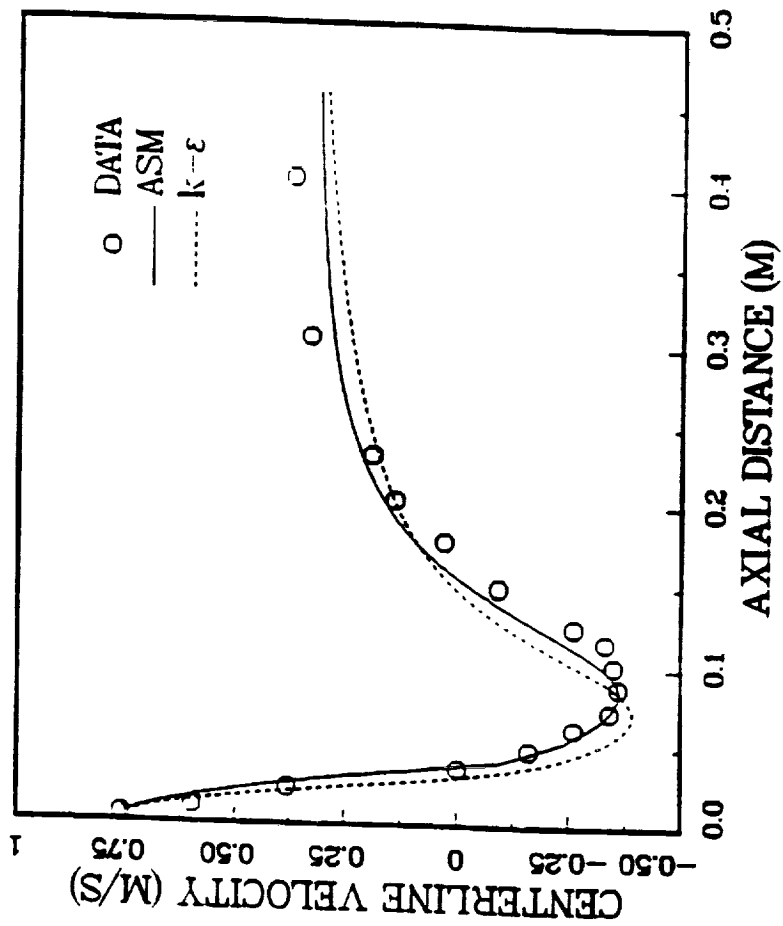


Fig. 6 Decay of mean axial centerline velocity.

# Confined Swirling Flows

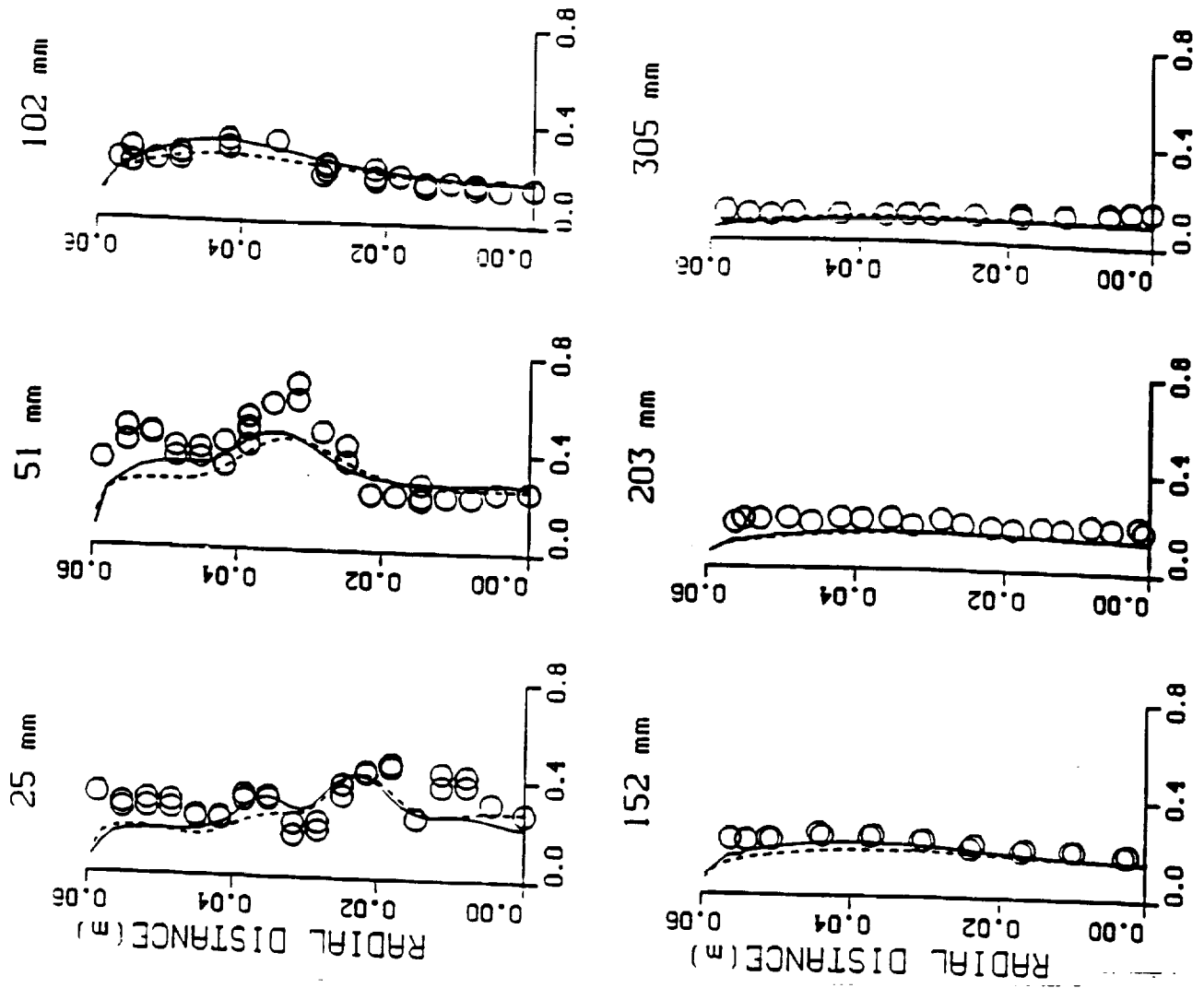


Fig. 8a Radial profiles of turbulent intensity ( $\sqrt{u'^2}$ )

# Confined Swirling Flows

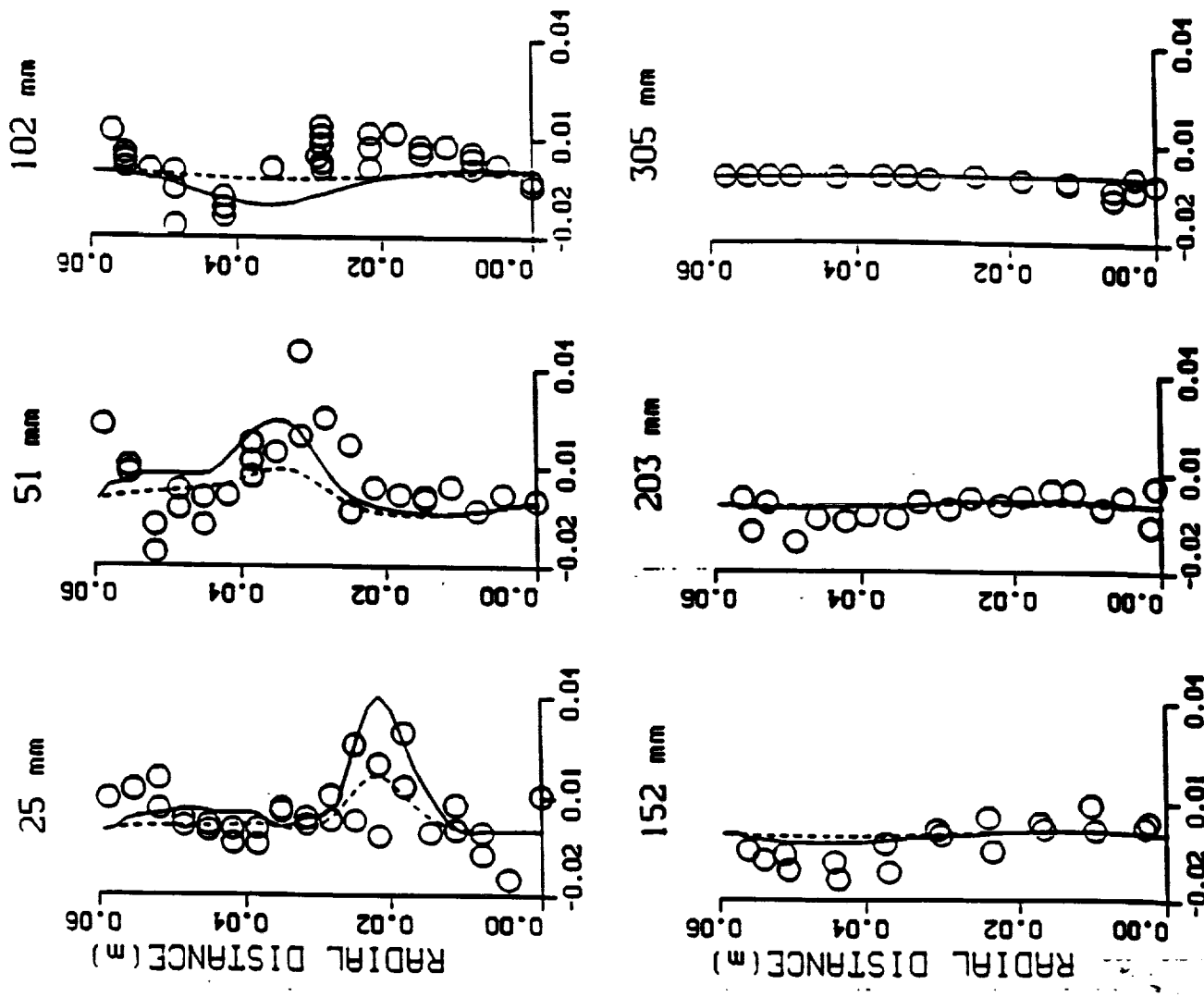


Fig. 8b Radial profiles of Reynolds stress ( $\overline{u'w'}$ )







## 8 - STEP REACTIONS

		A	N	E
M + O <sub>2</sub>	==== O + O	0.72000E+19	-1.0000	117908
M + H <sub>2</sub>	==== H + H	0.55000E+19	-1.0000	103298
M + H <sub>2</sub> O	==== H + OH	0.52000E+22	-1.5000	118000
O + H	==== OH	0.71000E+19	-1.0000	0.
H <sub>2</sub> O + OH	==== H <sub>2</sub> O + H	0.58000E+14	0.0000	18000
H <sub>2</sub> + OH	==== H <sub>2</sub> O + H	0.20000E+14	0.0000	5166
O <sub>2</sub> + H	==== OH + O	0.22000E+15	0.0000	16800
H <sub>2</sub> + O	==== OH + H	0.75000E+14	0.0000	11099

$$k = AT^N \exp(-E/RT)$$

with k in cm<sup>3</sup> · mole<sup>-1</sup> · s<sup>-1</sup> and E in cal · mole<sup>-1</sup>

From R.C.Rogers and Chinitz, ' Using a global hydrogen-air model in turbulent flow calculations ', AIAA J., vol. 21, pp. 586-592, 1983.

# SSME Nozzles

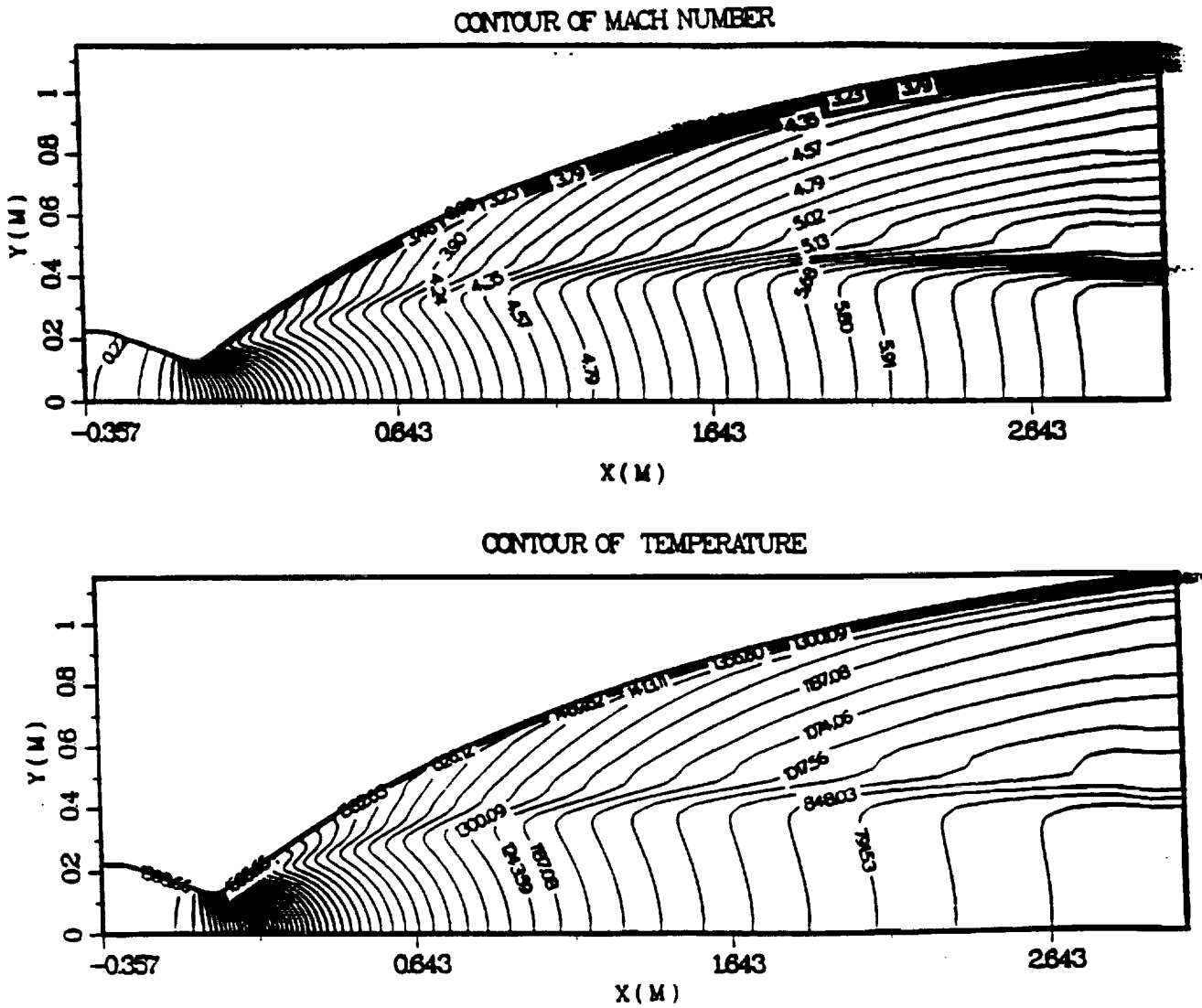


Figure A.5 Sample SSME Nozzle Flow Inputs and Results --- Turbulent,  
 8-Step Kinetics.  
 ISP = 452.78 sec. Exp. ISP=453.3sec

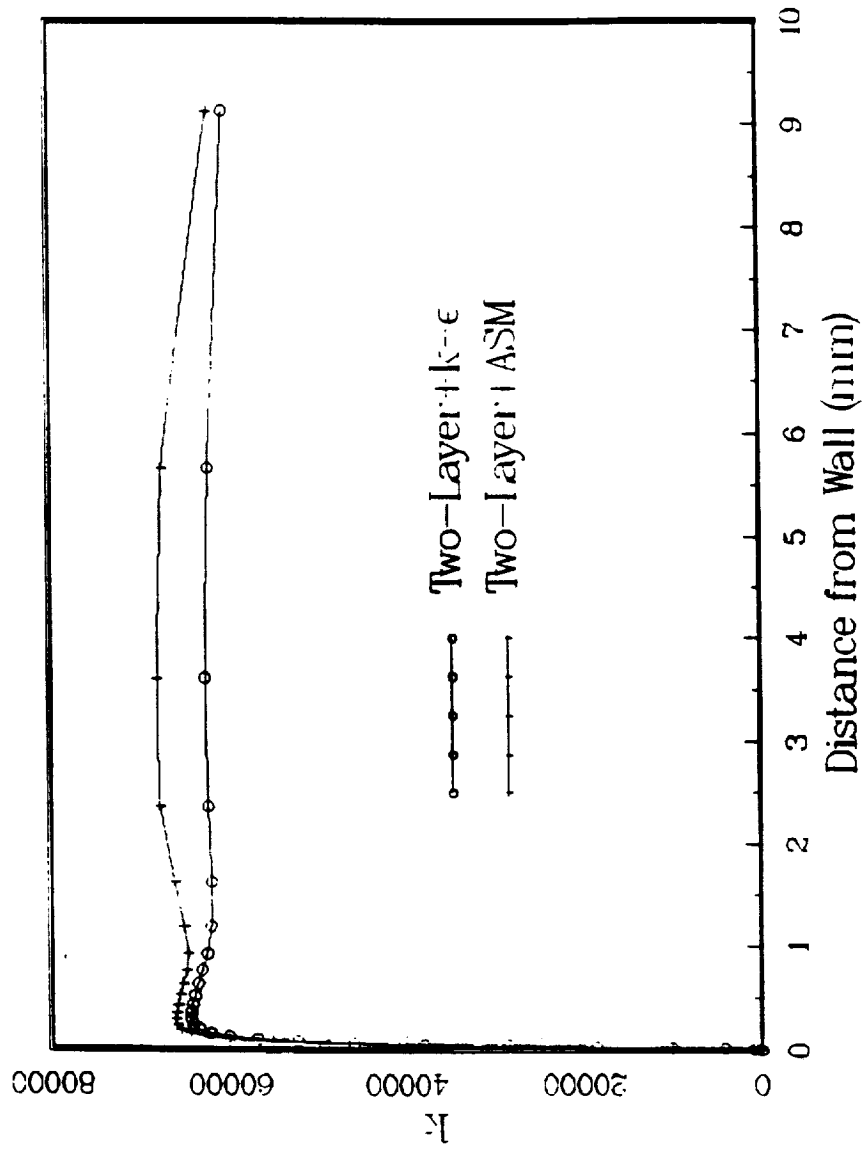


Fig. 11 Behavior of near wall TKE at nozzle exit.

# Testbed Engine 3001 Summary Review Combustion Devices (Tests 020-028)



National Aeronautics and  
Space Administration

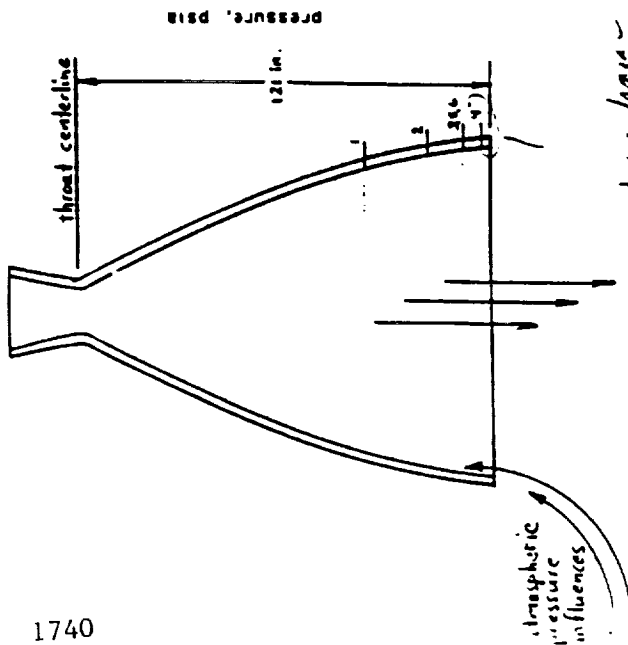
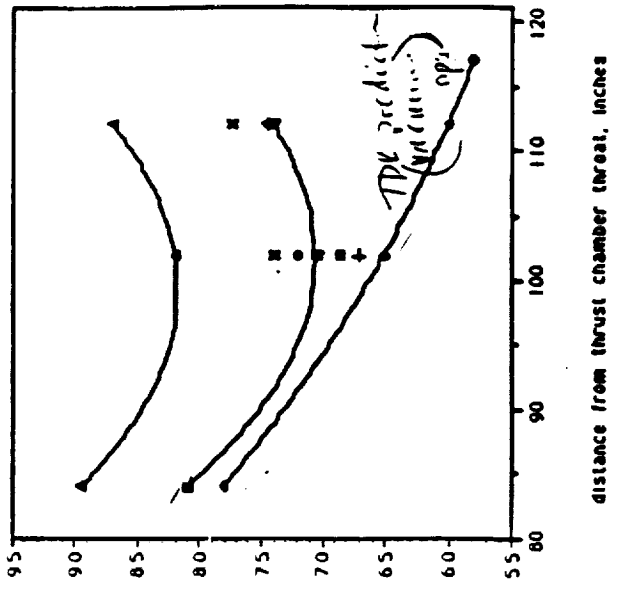
## Nozzle Wall Static Pressure

*Per View:*  
Increase associated  
by sea level  
not expected at  
vacuum conditions

To new CFD code  
to compare

- TDK 100E AB
- 100E-021 AB
- ▲ 100E-023 AB
- 100E-025 AB
- † 100E-026 AB
- 100E-027 AB
- 100E-028 AB

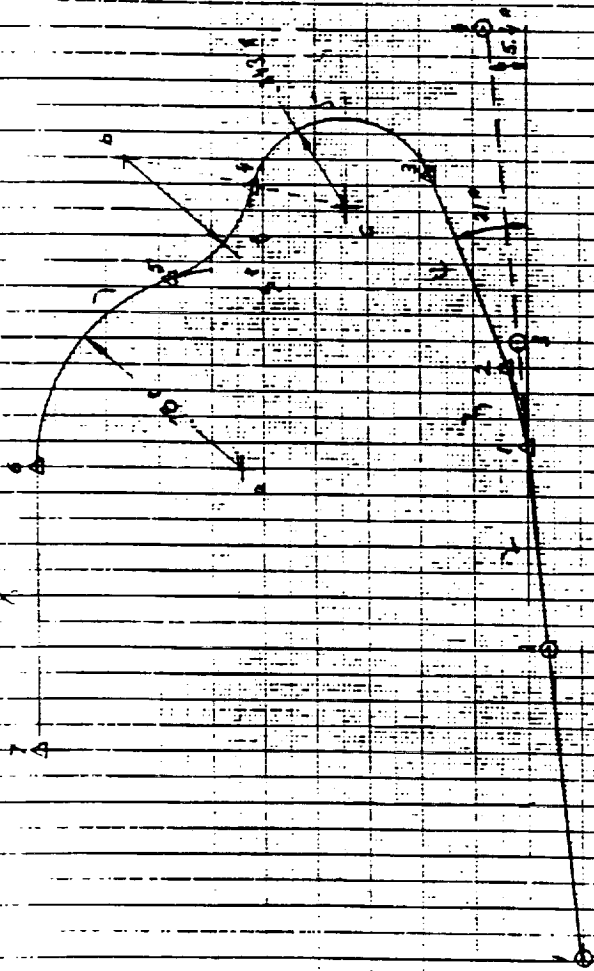
The location  
where P  
increases  
is a sharp  
It is further  
upstream than  
expected



new pressure  
new test  
No data available

SSME Wall Contour at Exit

$R_H = 5.1527 \text{ [in]}$



X [in]	Y [in]	R/R <sub>H</sub>
117	45.000	0.77325
118	45.175	0.77616
119	45.347	0.77924
120	45.518	0.78249
121	45.686	0.78591
122	45.851	0.78950
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Fig. 12, SSME wall contour and geometry at exit

SSME OUTLET  
GRID LINE

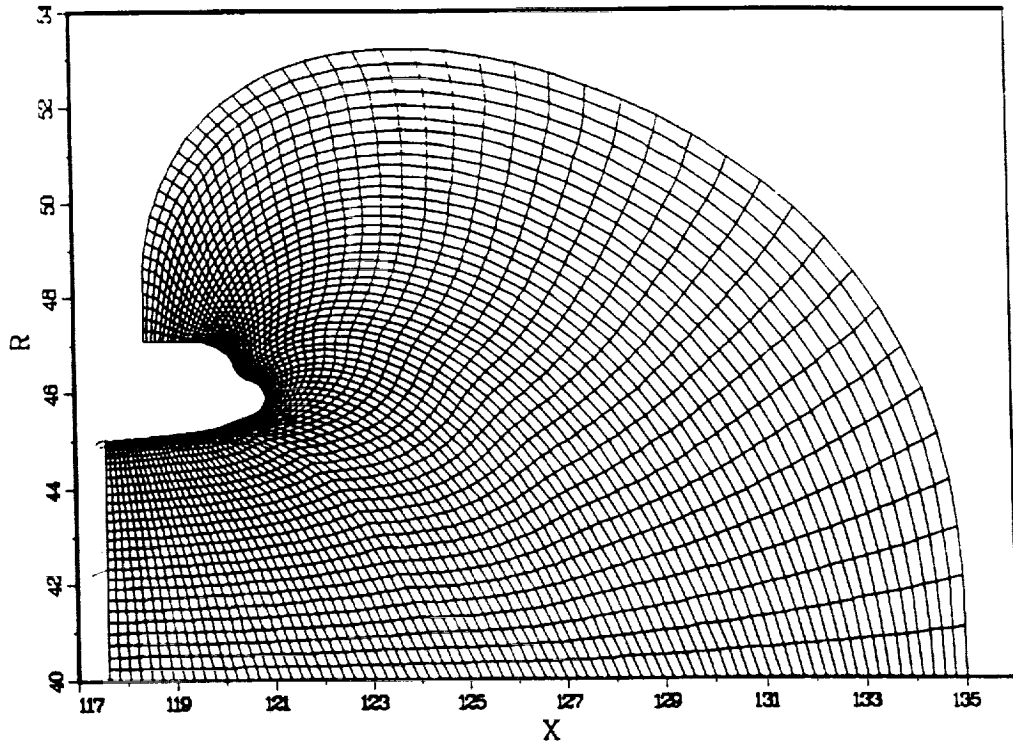


Fig. 13(a), Grid configurations for SSME nozzle exit manifold

GRID LINE

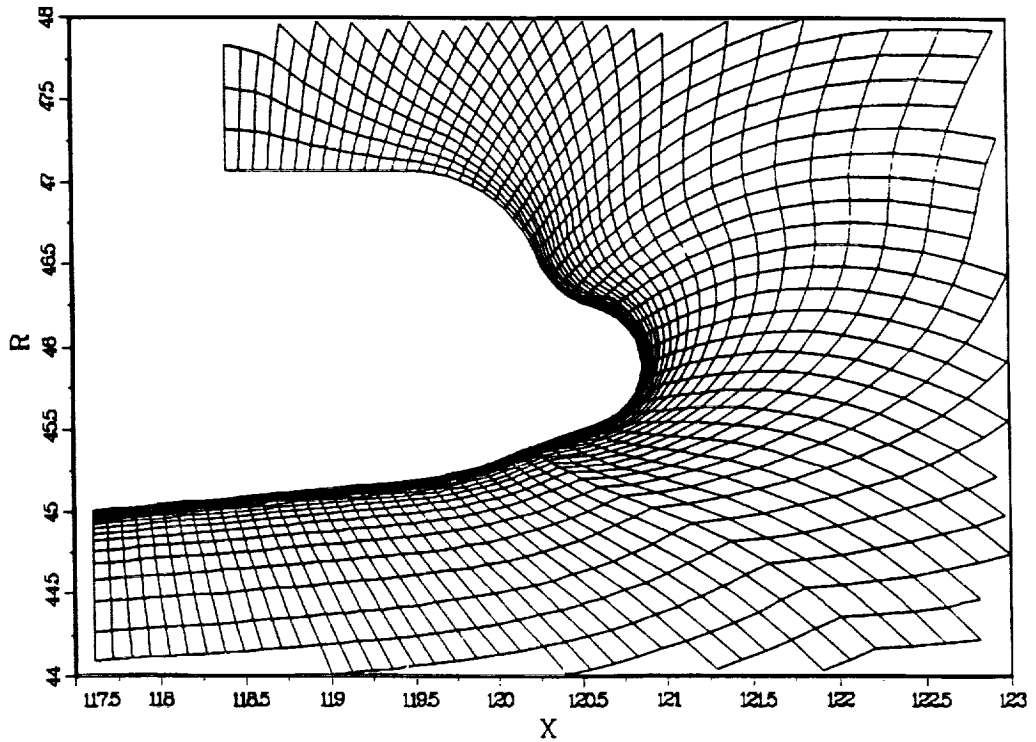
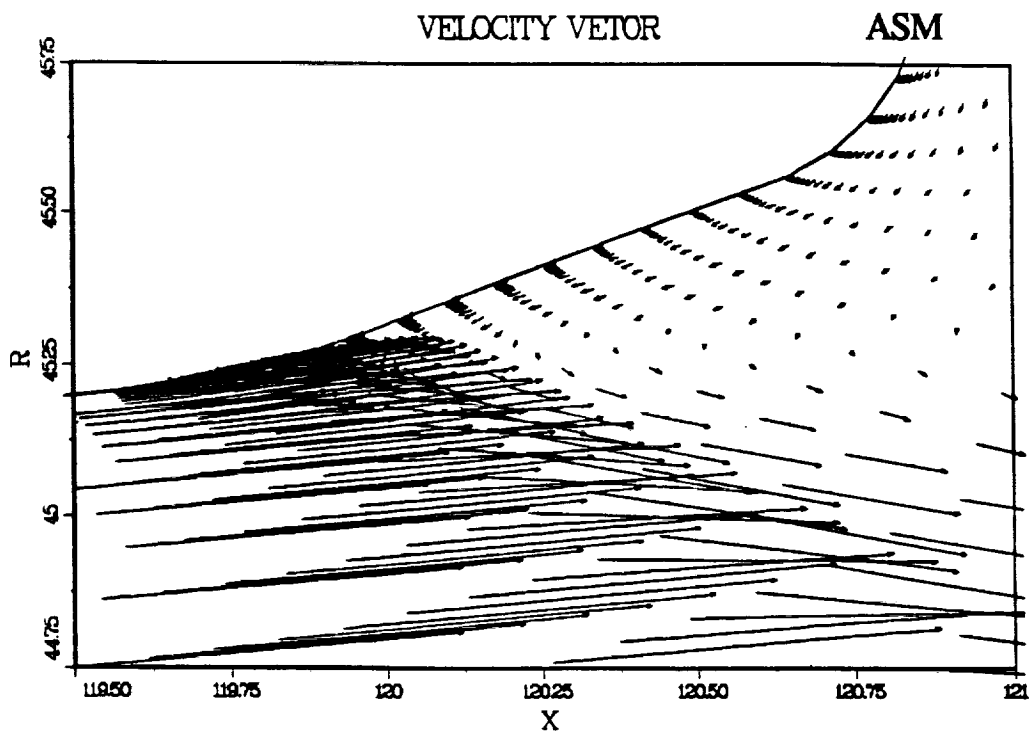
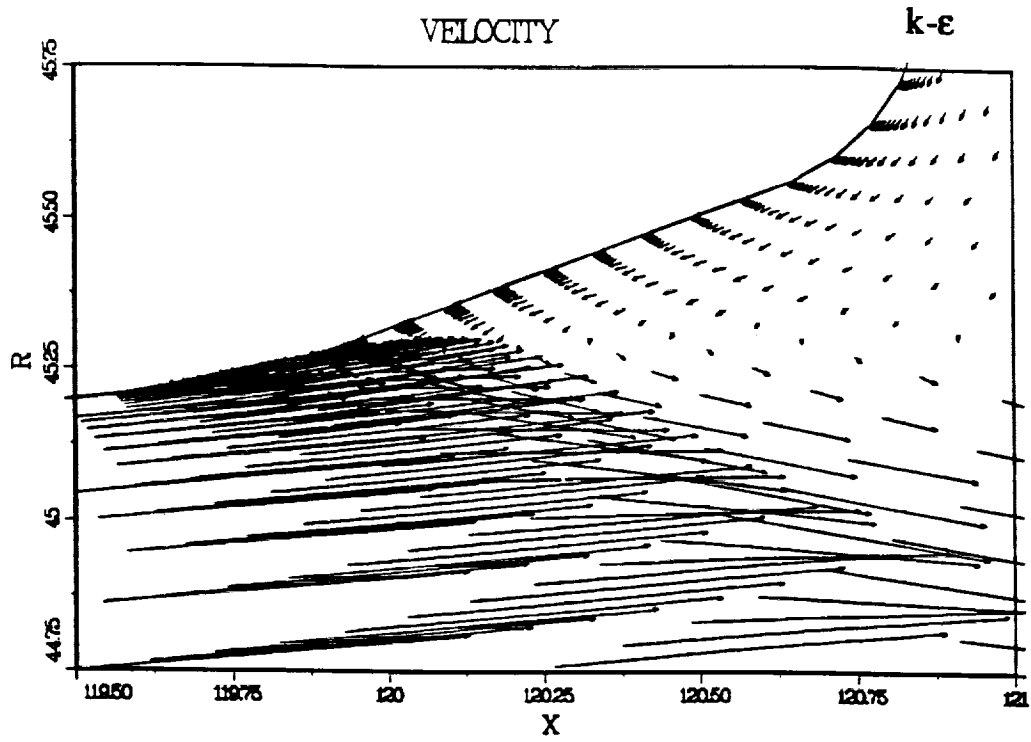


Fig. 13(b), Close-up grids for Figure 13(a)







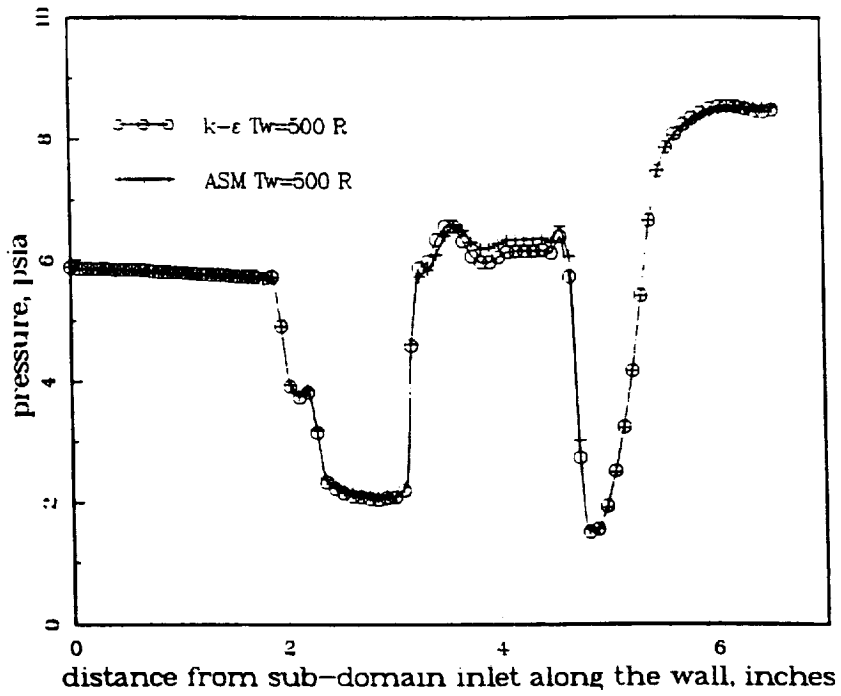


Fig. 17(a), Pressure levels along the wall near the nozzle exit using the ASM and  $k-\epsilon$  models

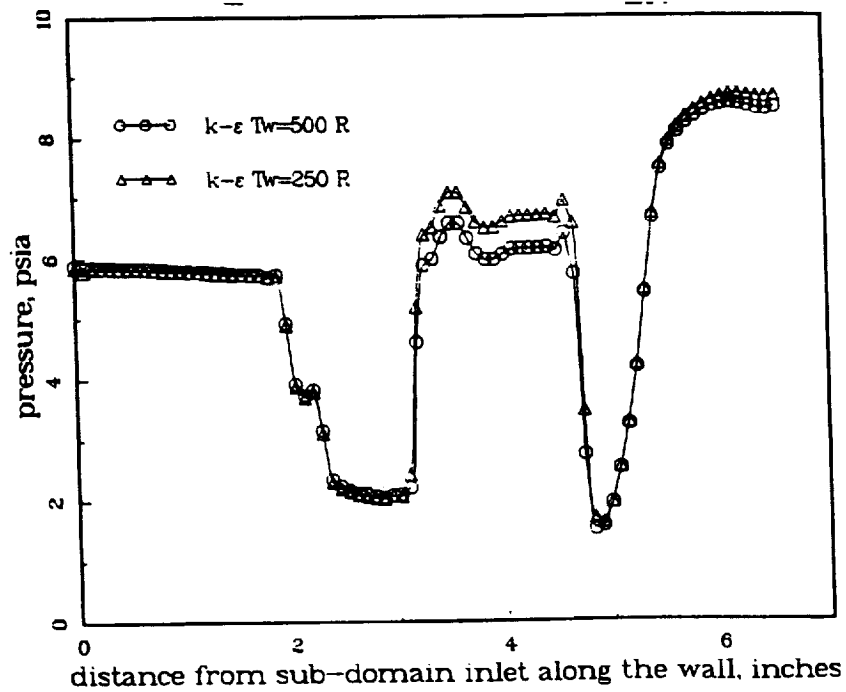


Fig. 17(b), Effects of wall temperature on the wall pressure

SSME OUTLET  
CONTOUR OF PRSSURE

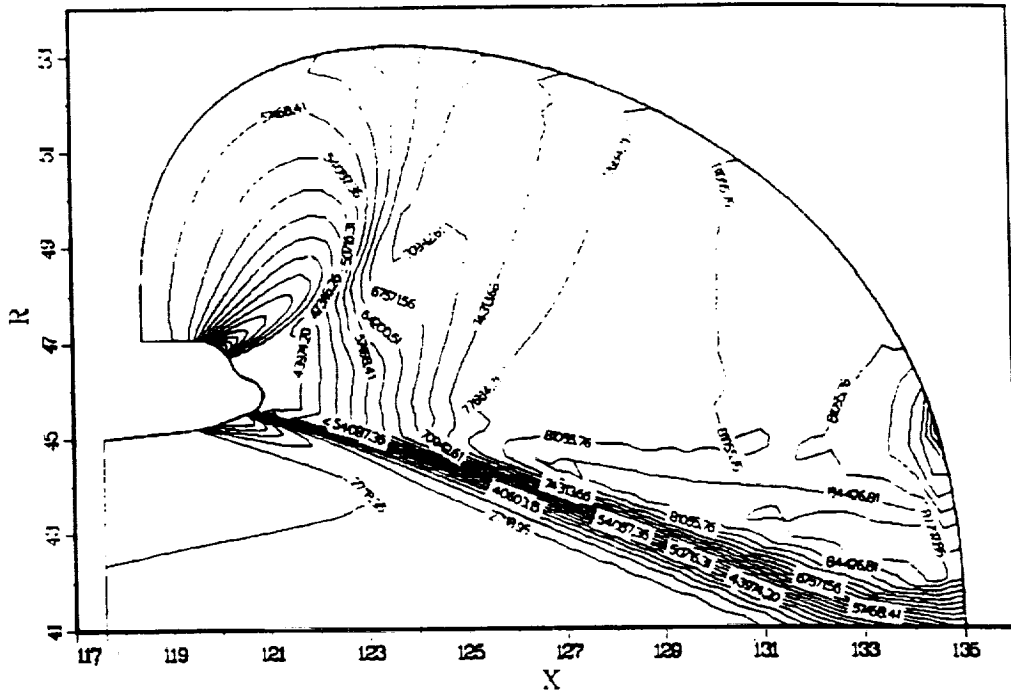


Fig. 18, Contour of pressure using 75% of the chamber pressure level

SSME OUTLET  
CONTOUR OF MACH NUMBER

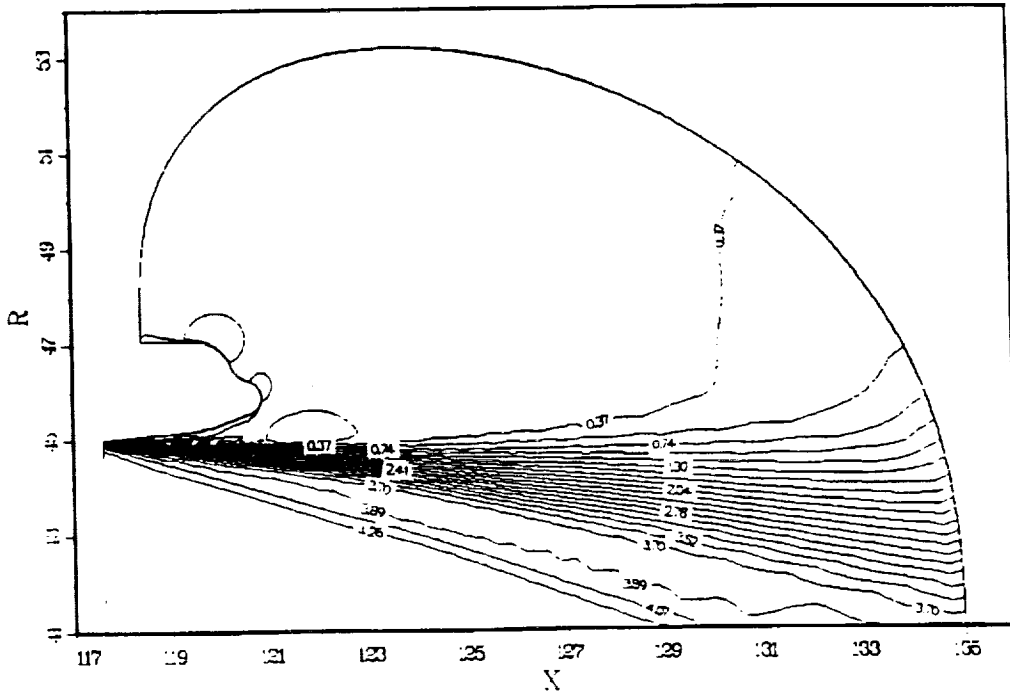


Fig. 19, Laminar flow calculations of the SSME exit flow

## Summaries

- The Algebraic Stress Model Removes the Isotropic Turbulence Assumption for the Eddy Viscosity Type Models
- Improved On the Reynolds Stresses Predictions
- The ASM Does Not Improve Too Much On SSME Nozzle & Outlet Flows
  - RSM
  - Other Mechanisms
  - Shock- Boundary Layer Interactions
  - Entrainment Issues
- 3-D Calculations Are Desirable

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