

# GRADUATE AERONAUTICAL LABORATORIES CALIFORNIA INSTITUTE OF TECHNOLOGY

LITERATURE SURVEY  
on the  
BUCKLING OF CONICAL SHELLS

by Shigeo Kobayashi

January 1966 Grant No. NsG-18-59

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California Institute of Technology

Pasadena, California

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By Shigeo Kobayashi<sup>\*</sup>

California Institute of Technology

SUMMARY

In order to prepare a comprehensive review of the state of the art of conical shell instability this literature survey was made. The bibliography is arranged in alphabetical form and is then categorized according to loading conditions. Certain key equations used by the authors are also indicated.

INTRODUCTION

Prior to setting up a research program on the buckling load of conical shells, an extensive literature search was made. Eighty-three (83) of these reports have been reviewed and key information from each has been tabulated. The remaining thirty (30) were not available at the time of writing this report but will be reviewed when they can be obtained. This survey is presented in the hope that it will aid others contemplating research on conical shell stability to find these reports of interest to them.

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\* Assistant Professor, University of Tokyo; Visiting Research Fellow, California Institute of Technology.

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\* Total number of references.

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5.2	Thermal Buckling - Experimental	1
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AXIAL COMPRESSION - ANALYSIS - 1

Bibliography No.	51)	20)	35)	41)
Author	Shtayerman	Mushtari and Sachenkov	Sachenkov	Seide
Periodical	1936, Non-Linear Theory 52	(PMM 1954) (NACA TM 1433, 1958)	(1955) Non-linear 56	J. Appl. Mech. 1956
Load; including internal pressure or not	no	no	no	no
Shell thickness; constant or not	constant	constant	constant	constant
Buckling stress or Post-buckling behavior	buckling stress	buckling stress	post-buckling behavior	buckling stress
Buckling mode; axisymmetrical or unsymmetrical or both	axisymmetrical	unsymmetrical	unsymmetrical	axisymmetrical
Differential equation (Strain displacement relation)	shallow shell approximation	shallow shell approximation	shallow shell approximation	Timoschenko - axisymmetrical (They are the same as those of shallow shell approximation)
Boundary Condition	$w=0, M_x=0, T_x=0, \epsilon_y=0$	$w=0, M_x=0, T_x=0, \epsilon_y=0$	not considered	$u \sin \alpha - w \cos \alpha = 0$ $M_x = 0$
Solution; exact or approximate	Galerkin Method ( $w$ is assumed, $F$ is obtained)	Energy Method	Exact solution (but buckling stress is not obtained exactly)	
Remarks	$w = A_0 X^{\frac{1-\gamma}{2}} \sin(m \log \frac{x}{x_0}) \cos nt$	Approximate treatment alone which leads the equations to those of cylindrical shell	Result;	$\frac{P}{P_{cyl} \cos^2 \alpha} = 1$

AXIAL COMPRESSION - ANALYSIS - 2

14)	45)	7)	36)	46)
Lackman and Penzien	Seide	Eringen	Schnell	Seide
J. Appl. Mech. 1960	J. Appl. Mech. 1961	Gen. Tech. Corp. Rep., 1959	ZAMM 1961	Proc. 4th U.S. Nat'l Congress. Appl. Mech. 1962
	no	no and internal pressure	internal pressure	internal pressure
	constant	constant	constant	constant
	buckling stress	post-buckling behavior	buckling stress	buckling stress
	axisymmetrical	unsymmetrical and axisymmetrical	axisymmetrical	axisymmetrical
			shallow shell approximation	shallow shell approximation
		not considered	(A) Clamped;	(A) Clamped;
			$\epsilon_{\theta} = \frac{dw}{dx} = 0$	$\epsilon_{\theta} = \frac{dw}{dx} = 0$
			(B) Simply supported;	(B) Simply supported;
			$\epsilon_{\theta} = 0$	$\epsilon_{\theta} = 0$
		Energy Method	Fourier series; using special types of Galerkin Method	Fourier series; using special types of Galerkin Method
Discussion on Seide's paper Ref. 41)	Reply to Lackman and Penzien's discussion paper, Ref. 14)	Seide's result is quoted	$w = A \sin Kx \cos \eta \theta + B \cos 2Kx + C$	(A) Clamped; $\frac{dw}{dz} = \sum_{m=1}^{\infty} a_m \sin \frac{m\pi z}{z_0}$ (z = log $\frac{x}{x_1}$ , $(z_0 = \log \frac{x_2}{x_1})$ )
				(B) Simply supported; $\frac{dw}{dz} = \sum_{m=1}^{\infty} a_m \cos \frac{m\pi z}{z_0}$



AXIAL COMPRESSION - ANALYSIS - 3

57)	33)	37)	38)	79)
Singer	Ricardo, Marreco and Csokuyai	Schnell	Schnell	Singer
TAE Rep. 19, Sec. 4, 1962	J. M. E. S. 1963	Z. Flugwiss, 1962, Teil I	Z. Flugwiss, 1962, Teil II	AIAA Journal, 1965
no	no	no and internal pressure	no and internal pressure	
constant	constant	constant	constant	
buckling stress		buckling stress	post-buckling behavior	
unsymmetrical	unsymmetrical	unsymmetrical	unsymmetrical and axisymmetrical	
shallow shell approximation		shallow shell approximation	shallow shell approximation	
Simply supported;			not considered	
$w=0, M_x=0 (v=0, T_x=0$ are not satisfied) $u, v$ are determined from the differential equations)				
Galerkin Method for 3rd eq. ( $w$ : series expansion)		Energy Method	Energy Method	
	Discussion like that of Yoshimura	$w = A \sin \chi z \cos \gamma \theta$	$w = s, (\bar{A} \sin \chi z \cos \gamma \theta + \bar{B} \cos 2 \chi z + \bar{C})$	Same as Ref. 57)
	$S = \sum_{n=1}^N \sum_{m=1}^M C_n \chi \sin n t \theta$			
	$\beta = \frac{\log \chi z}{\pi}$			
	satisfy the boundary conditions $w=M_x=0$ )			

AXIAL COMPRESSION - EXPERIMENTAL - I

Bibliography No.	16)	15)	14)	17)
Author	Lundquist and Schuette	Lofblad	Lackman and Penzien	Lutwak
Periodical	NACA WR, 1942	MIT TR 1959	J. Appl. Mech., 1960	UCLA, MS Thesis, 1960
Load; including internal pressure or not	no	internal pressure	no	no
Specimen material	17 ST	7075 ST-6	Nickel	Mylar
Use of test specimens	one time	several times	one time	several times
Seam	Bolted with butt strap	1/2 in. lap joint, bonded with Epon IV	Seamless, made by electroplating	1.25 in. lap joint, fastened with double-backed adhesive scotch tape
Boundary support	Clamped to end bulkhead with steel band	Fixed with Cerrobend		Clamped with tapered blocks
Semi-vertex angle $\alpha$	11.3°	2.9°	20°, 40°	10°, 20°, 30°, 45°, 60°
Radius of Bottom Circle	R - inches 7.5	5		5
Shell Thickness, h - inch	0.011	0.004	0.005	0.002
Vertical Height L inches	7.5	20		4, 8
Internal Pressure parameter $\bar{p} = \frac{P}{E} \left( \frac{R}{t} \right)^2$		0	1.09	
Comparison with theory $F_{cr}/P_{cl}$	0.38	0.53	0.25	0.5

AXIAL COMPRESSION - EXPERIMENTAL - 2

5)	71)	39)	76)	81)
Brown and Rea	Weingarten, Morgan and Seide	Schnell and Schiffner	Berkovito and Singer	Weingarten, Morgan, and Seide
Air Univ. MS Thesis, 1960	STL, TR, 1960	DVL, 1962	7th Israel Am. Conf., 1965	AIAA Jour., 1965
Internal pressure	no internal pressure and internal pressure	no internal pressure and internal pressure	no	
17-7 PH	Mylar (steel for comparison)	Mylar		
Several times (in case of internal pressure)	Many times (changing pressure)	Many times (changing pressure)		
0.5 in. lap joint, bonded with adhesive and spot welded	0.75 in. lap joint, bonded with eccobond			
	fixed with Cerrolow			
	0° 10° 20° 30° 45° 60°	0°, 10°, 20°, 30°		
5		200 mm		
0.005	0.002 ~ 0.01	0.125mm, 0.255mm		
10		173 mm		
0.07 ~ 0.96	0 ~ 5	0 ~ 1.5		
	no internal pressure			
	0.26 ~ 0.81			
	internal pressure	0.5 ~ 1.2		
	~ 1.55			
	Effect of eccentricity by seam is considered in determination of buckling load	Buckling load is high; $P_{cr} / P_{cl} = 0.601 \sim 0.965$		Same as Ref. 71)

AXIAL COMPRESSION - EXPERIMENTAL - 3

83)

Weingarten, Morgan  
and Seide

AIAA Journ. 1965

Same as Ref. 71)

EXTERNAL PRESSURE - ANALYSIS - I

Bibliography No.	68), 69)	26)	25)	3), 4)
Author	Tokagawa	Pflüger	Niordson	Býlaard
Periodical	JSNA (1932 1940)	Ing. Arch. 1937	Tr. Roy Inst. Tech. Sweden, 1947	Bell Air Co. T. R. 1953 and NASA TN D-1510
Complete cone with apex or trun- cated cone	complete and truncated	complete	truncated	complete and truncated
Hydrostatic pressure alone or + axial com- pression, or other	hydrostatic	(1) hydrostatic (2) dead weight	hydrostatic	hydrostatic
Pressure, uniform or not	uniform	uniform	uniform	uniform and non- uniform
Shell thickness; constant or not	constant	linear- variation	constant	constant
Small deflection theory or large deflection theory		small deflection theory	small deflection theory	small deflection theory
Differential equation		Pflüger's equation	Niordson's equation	
Exact solution or approximate solution		exact solution of differential equation	Energy Method	Energy Method
Satisfy or not boundary condition		satisfy B. C. at apex but do not satisfy B. C. at lower end	not considered	B. C. is simply sup- ported. It is approx- imately satisfied
Include or not effect of prebuckling de- formation		not considered	not considered	not considered
Remarks	A proposal of equivalent cylinder radius and design formula, and com- parison with experi- ment.	Solution which satisfy the differential equation; $u = Ap^n \cos \lambda \varphi$ $u = Bp^n \sin \lambda \varphi$ $w = Cp^n \cos \lambda \varphi; n=2$ good	Assumed displacement; $u = A \cos n \theta \cos \lambda \xi$ $u = B \sin n \theta \sin \lambda \xi$ $w = C \cos n \theta \sin \lambda \xi$ simple but not too good	$w = C \tau^2 \sin \lambda x \cos n \theta$ (select suitable length $\lambda$ )

EXTERNAL PRESSURE - ANALYSIS - 2

20)	9)	34)	67)	30), 29)
Mustari and Sachenkov	Grigolyuk	Sachenkov	Taylor	Radkowski
PMM 1954 (NACA TM 1433)	1954, Translation in ASTIA	1956 (Ref. 19- 54)	3rd Midwestern, 1957	Prof. 3rd N. C. Appl. Mech. 1958 Watertown Lab. 1958
truncated	complete	complete and truncated	complete	truncated (complete)
hydrostatic and external pressure + axial compression	hydrostatic	hydrostatic	hydrostatic	hydrostatic
uniform	uniform	uniform	uniform	uniform
constant	constant	variable thickness linear	constant	constant
small deflection theory	small deflection theory	small deflection theory	small deflection theory	small deflection theory
Donnell type	was not written	Donnell type	was not written, but corresponds to Pflüger type	
Galerkin Method	Energy Method	Exact solution of differential equation	Energy Method	Energy Method
Simply supported $w=0, M_x=0, T_x=0, \epsilon_\theta=0$ satisfy completely	Satisfy B. C. at apex and at lower end	He considered a long shell. Did not satisfy boundary condition	Satisfy B. C. at apex and $u \sin \alpha + w \cos \alpha = 0, U=0$ at lower end	
not considered	not considered	not considered	not considered	not considered
Assumed deflection	Assumed deflection			Assumed displacement;
$w = A_0 e^{\gamma z} \sin m z \cos n \varphi$	$w = A_1 r^2 \sin \frac{\pi z}{b} \sin n \varphi$	$u = 0$ $U = B x^3 (L-x) \sin n \theta$ $w = C x^3 (L-x) \cos n \theta$		$u = A_1 \cos n \theta \cos \frac{\pi z}{b}$ $U = B_1 \sin n \theta \sin \frac{\pi z}{b}$ $w = C_1 \cos n \theta \sin \frac{\pi z}{b}$
$(z = \frac{x}{X_0})$		Expression of strain-displacement relation in large deflection theory is shown		

EXTERNAL PRESSURE - ANALYSIS - 3

11)	44)	28)	54)	18)
Hoff and Singer (Proc. IUTAM, 1959)	Seide Proc. IUTAM, 1959	Pittner and Morton Lockheed Rep., 1961	Singer J. Mech. Eng. Sci. 1961	Mescall WAL. TR. 1961
truncated	truncated and complete	truncated	truncated	truncated and complete
hydrostatic	hydrostatic	hydrostatic and external pressure + axial compression	(1) hydrostatic (2) external pressure (3) external pressure + axial compression	hydrostatic
uniform	uniform	uniform	uniform and non-uniform	uniform
constant	constant	constant	constant	constant
small deflection theory	small deflection theory	small deflection theory	small deflection theory	(large deflection theory)
Hoff's equation	Seide's equation (Donnell type)	was not written	Donnell type	shallow shell assumption
Galerkin Method	Energy Method	Energy Method	Galerkin Method	Energy Method
$w=0, M_x=0$	satisfy $U \sin \alpha - w \cos \alpha = 0$	satisfy $U=0, w=0$ AND $M_x=0$	satisfy $w=0$ and $M_x=0$	
does not consider condition for $U$ and $U$	and $U=0$ at both ends	does not consider condition of	does not consider conditions for $U$ and $U$	
not considered	not considered	not considered	not considered	includes the effect of prebuckling deformation
Assumed displacement $U = \sum_{n=1}^N A_n X^S \sin t \phi$	Assumed displacement $U = \sum_{m=1}^M C_m \sin(m \frac{Z}{L_0}) \cos m \theta$	Assumed displacement $U = C_1 e^{\mu Z} \cos \lambda Z \cos \eta S$	Assumed Displacement $U = \sum_{n=1}^N A_n X^S \sin t \phi$	$w = w_0 + A \left[ \frac{2\beta}{1+\beta^2} \left( \frac{r}{r_0} \right)^2 + (1-\beta)^2 \right] \times \cos(ar+b) \times \cos m \theta$
Correction of mistake About 12 terms are shown in Ref. 52 required	$e^{\frac{Z}{L_0}}$ $e^{\frac{Z}{L_0}}$ $e^{\frac{Z}{L_0}}$	$U = C_2 e^{\mu Z} \sin \lambda Z \sin \eta S$ $w = C_3 e^{\mu Z} \sin \lambda Z \cos \eta S$ ( $e^{\frac{Z}{L_0}}$ $e^{\frac{Z}{L_0}}$ )	$U = \sum_{n=1}^M B_n X^S \cos t \phi$ $w = \sum_{m=1}^M C_m X^S \sin t \phi$	etc.

EXTERNAL PRESSURE - ANALYSIS - 4

22)	Myursepp	24)	Newman and Reiss	53)	Singer	62)	Singer	73)	Williams
	Iz Ak Ne. 1961 (FTDIT 62-342 1962)		NASA TN D 1510, 1962		J. Appl. Mech., 1962		AIAA Jour. 1963		(Navy Rep. 1952)
truncated		complete and truncated		truncated					
external pressure (= cone alone)		external pressure (= cone alone)		hydrostatic					
linear variation		uniform		uniform					
constant		constant		constant					
small deflection theory		large deflection theory		small deflection theory					
		axisymmetrical deformation in large deflection theory		Donnell type					
Eigen value problem of diff. eq. Pertur- bation method		Finite difference iteration		Energy Method					
satisfy		simply supported satisfy		$w=0$ and $v=0$					
not considered		not considered		not considered					
		Axisymmetrical buckling of shallow cap.		Calculation of the effect of axial con- strain		Continuation of Ref. 54). Discussion on equivalent cylindrical shell		Numerical table of Niordson's theoret- ical result 25)	
				$u = A \sin t \phi \cos \lambda \xi$					
				$v = B \cos t \phi \sin \lambda \xi$					
				$w = C \sin t \phi \sin \lambda \xi$					





EXTERNAL PRESSURE - EXPERIMENTAL - I

Bibliography No.	68)	10)	12)	40)
Author	Tokugawa	Harris and Leyland	Jordan	Schroeder and Kusterer
Periodical	JSNA, 1932	Tr. In. Chem. Eng. 1952	U. Alabama, 1955 ASTIA	J. Appl. Mech. 1963 Aircraft Armaments, 1958
Complete cone with apex or truncated cone	complete and truncated	complete	complete	complete
Load	Hydrostatic - uniform	Hydrostatic - uniform	Hydrostatic - uniform	Hydrostatic - uniform
Specimen material	brass	brass, copper, monel, steel	24S-T3, 61S-T6	yellow brass
Use of test specimen	one time	one time	one time	one time
Seam	seam	seam 1/8 in., soldered	seam welded	seam soldered
Boundary support	thick end plate	clamp the edge like hat		steel ring
Semi-vertex angle $\alpha$	$3^\circ \sim 34^\circ$	$20^\circ \sim 75^\circ$	$10^\circ, 15^\circ, 20^\circ, 25^\circ$	$25^\circ, 15^\circ, 10^\circ$
Radius of bottom circle	R inches 50mm	1.3 ~ 6	6	2
Shell thickness h inches	0.5mm	0.013 ~ 0.0705	0.04, 0.05, 0.06, 0.07, 0.08	0.010
Vertical height L	10mm ~ 180mm			2R, 4R, 6R
No. of test specimens or No. of test points	test specimen: 50	test specimen kinds = 20	test specimen numbers = 52	test specimen: 10 (other reinforced specimen = 2)
Loading method	hydraulic pressure (water)	hydraulic pressure (oil)	hydraulic (oil)	hydraulic (oil)
	In some cases plastic buckling is observed	experimental formula is derived		

EXTERNAL PRESSURE - EXPERIMENTAL - 2

32)	71)	56)	60), 57)
Raetz	Weingarten, Morgan and Seide	Singer and Eckstein	Singer and Eckstein
David Taylor M.B. Rep., 1960	STL Final Report, 1960	Bul. Res. Con. Israel, 1962	Proc. 5 Israel, 1963 (TAE Rep. 19)
truncated	truncated	truncated	truncated
hydrostatic - uniform	hydrostatic - uniform *external + axial compression	hydrostatic - uniform	hydrostatic - uniform
steel	Mylar (steel)	Al. alloy, AG 5- X516	stainless steel (18-8) 2024-T3
seam welded	many times (changing pressure)	one time	steel; one time Al. alloy; many times
stiffening ring at both ends	seam (Eccobond)	seam butt welded	St.; bull weld Al.; lap bond
30°, 45°, 60°	fix by Cerrolow	simply supported	two types: simply supported clamped
13.5	0°, 10°, 20°, 30°, 45°, 60° (steel; 75°)	10°, 20°, 30°	30°, 40°
0.1	4	317.5 mm	317.5 mm
radius of upper circle = 8 in.	0.01 ~ 0.002 (steel; 0.01 , 0.02 )	0.5 mm, 0.6 mm, 0.8 mm, 1.2 mm, 1.6 mm	0.4 mm, 0.6 mm, 0.8 mm, 1.0 mm
test specimen = 6	test specimen; kinds = 30 numbers = 71	radius of upper circle = 50 mm., 127 mm.	kinds of test specimen = 10
stress distribution was measured before collapse	evacuate inside air *to obtain inter- action curve	hydraulic (oil)	hydraulic (oil)
		improvement of experiment in previous paper 56)	



TORSION - ANALYSIS - 1

Bibliography No.	27)	21)	47)	57)
Author	Pflüger	Mushtari	Seide	Singer and Baruch
Periodical	Ing. Arch. 1942	PMM 1943, Ref. 19)	J. Appl. Mech. 1962	TAE Rep. 19, Sec. 1, 1962
complete cone with apex or truncated cone	truncated		truncated	truncated
Load	torsion	torsion	torsion	torsion and torsion + external pressure*
shell thickness; constant or not	constant	constant	constant	constant
small deflection theory or large deflection theory	small deflection theory	small deflection theory	small deflection theory	small deflection theory
differential equation	Pflüger's energy expression	Donnell's equation	Seide's equation (Donnell type)	Seide's equation (Donnell type)
exact solution or approximate solution	Energy Method	Energy Method	Energy Method	Galerkin Method
boundary condition; satisfied or not	not considered	not considered	simply supported satisfied	modified simply supported satisfied
effect of prebuckling deformation included or not	not considered	not considered	not considered	not considered
Remarks	Assumed displacement: $u = A \rho^2 \sin[\lambda z - m(\rho - \rho_2) \sin \alpha]$	Assumed displacement: $w = -2 C e^{\frac{z}{l}} \sin(\mu_1 z + \eta y)$	Method of analysis is the same as his paper Ref. 44).	Method of analysis is the same as his paper Ref. 54).
	$v = B \rho^2 \sin[ \quad ]$	$u = C e^{\frac{z}{l}} \{ \sin(\mu_1 z + \eta y) \}$		*interaction curve is obtained
	$w = C \rho^2 \cos[ \quad ]$	$v = C_2 e^{\frac{z}{l}} \{ \quad \}$		

TORSION - ANALYSIS - 2

61)	76)	
Singer	Berkovito and Singer	
TAE Rep. 30, Sec. 2, 1963	7th Israel Am. Conf., 1965	
truncated		
torsion + axial compression*		
constant		
small deflection theory		
Seide's equation (Donnell type)		
Galerkin Method		
modified simply supported satisfied		
not considered		
Method of analysis is the same as his paper Ref. 54)		
*interaction curve is obtained		

TORSION - EXPERIMENTAL - 1

Bibliography No.	16)	71)	60), 57)	61)
Author	Lundquist and Schutte	Weingarten, Morgan and Seide	Singer and Eckstein	Berkovito and Singer
Periodical	NACA WRL 1942	STL Final Report, 1960	Proc. 5 Israel Conf. 1963, TAE Rep. 19, Sec. 3	TAE Rep. 30, Sec. 3, 1963
complete cone with apex or truncated cone	truncated	truncated	truncated	truncated
Load	torsion	torsion	torsion and torsion + external pressure*	(1) torsion (2) axial compression (3) torsion + axial compression
specimen material	17 S-T	steel	2024S-T3, steel	2024S-T3, AG 5-X 516
use of test specimen	one time		2024S-T3; several times steel; one time	2024S-T3; several times
seam	seam bolted		seam: Al.: lap bond St.: butt weld	seam: Al.: lap bond AG 5-X516: butt weld
boundary support			simply supported	simply supported
semi-vertex angle $\alpha$		30°, 60°	30°, 40°	40°
radius of bottom circle R inches	7.5	10	317.5 mm	140 mm
shell thickness h inch	0.011, 0.016	0.01, 0.02	0.4 mm, 0.6 mm	2024S-T3: 0.4 mm AG5-X516: 0.53 mm
vertical height L	radius of upper end 6 in.			
No. of test specimens or No. of test points	No. of test specimen 5	Test specimen 10	Test specimen 20	Test specimen 20
	Comparison with the theory on cylinder	Comparison with the theory in Ref. 47)	Comparison with the thesis in Ref. 47) and 57)	Interaction curve between torsion and axial compression

\*interaction curve

TORSION - EXPERIMENTAL - 2

72)	76)	
Weingarten	Berkovito and Singer	
AIAA Jour. 1964	7th Israel Am. Conf. 1965	
truncated		
(1) torsion	(1) torsion	
(2) torsion + internal pressure	(2) compression	
	(3) torsion + compression	
Mylar		
many times		
seam 3/4 in. lap joint		
fixed to Al. end plates by Cerrobend		
20°, 30°, 45°, 60°		
5		
0.01 and 0.005		
Comparison with theory: torsion; Seide torsion + internal pressure; Singer		



ORTHOTROPIC - ANALYSIS - I

Bibliography No.	64)	55)	50)	58)	59)*
Author	Steyer and Zien	Singer	Serpico	Singer and Fersht-Scher	
Periodical	U. Alabama Rep. 1961	J. Appl. Mech. 1963 (TAE Rep. 18)	AIAA Jour., 1963	Aero Quart. 1964, (TAE Rep. 22)	
complete cone with apex or truncated cone	truncated		truncated	truncated	
Load	external pressure (+ axial compression)	equation alone	external pressure (+ axial compression)	external pressure	
orthotropic property: constant or not	constant	constant	constant	constant	
shell thickness: constant or not	linear variation		constant	constant	
small deflection theory or large deflection theory	small deflection theory	small deflection theory	constant	constant	
differential equation		small deflection theory		small deflection theory	
exact solution or approximate solution		Singer's equation (Donnell type)	Donnell type	Singer's equation	
boundary condition: satisfied or not			one term Rayleigh Ritz Method	Galerkin Method	
effect of prebuckling deformation: included or not	not considered	not considered	not considered	Modified Singer type	
Remarks		Derivation of Donnell type equation for orthotropic conical shell.		not considered	Application of Singer's method (Ref. 54)
					*NASA TN D-1510

ORTHOTROPIC - ANALYSIS - 2

2)	59)*	61), 63)	
Baruch and Singer	Singer and Fersht-Scher		
TAE Rep. 28, 1963	(TAE Rep. 30, Sec. 4, 1963) and (6th Am. Conf. 1964)		
truncated	truncated		
external pressure	torsion (+ external pressure)		
not constant, stiffened shell	constant		
constant	constant		
small deflection theory	small deflection theory		
Donnell type new equation	Singer's equation		
Galerkin Method	Galerkin Method		
Singer type, modified one	Singer type, modified one		
not considered	not considered		
*NASA TN D-1510	Application of Singer's Method (Ref. 54)		

ORTHOTROPIC - EXPERIMENTAL

Bibliography No.	61) 59)
Author	Singer, Eckstein, and Berkovito
Periodical	TAE Rep. 30, Sec. 1, 1963
complete cone with apex or truncated cone	truncated
Load	hydrostatic pressure
specimen material	AG5 - X516 (3)* SAE 1015 (4)* 2024 T3 (3)*
use of test specimen	one time
seam	AG5-X516 butt weld SAE 1015 lap bond 2024 T3
stringer, ring	AG5-X516 machining SAE 1015 bonding or 2024 T3 riveting
boundary support	
semi vertex angle $\alpha$	22°, 20°, 40° (3)* (4)* (3)*
radius of bottom circle R	140 mm, 317.5 mm
shell thickness $h$	0.60 mm 0.80 mm
vertical height L	
loading method	
	Including local buckling *number of test specimen

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