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A MONTE CARLO PROGRAM FOR TRANSMISSION  
PROBABILITY CALCULATIONS INCLUDING  
MASS MOTIONS

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TECHNICAL MEMORANDUM X-53386

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ABSTRACT

A Monte Carlo computer program which can be used to calculate the transmission probabilities for two cylindrical tubes in series is described, as well as the technique of adding directed mass motion to the random thermal motion of the molecules. The results for a simple straight cylindrical duct are compared to other solutions, and angle-of-attack effects are examined. Complex systems for which no adequate solutions have previously been found are easily analysed by this method. The Fortran instructions are listed along with a typical program solution.

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AERODYNAMICS DIVISION  
AERO-ASTRODYNAMICS LABORATORY

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A MONTE CARLO PROGRAM FOR TRANSMISSION PROBABILITY CALCULATIONS  
INCLUDING MASS MOTIONS

SUMMARY

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A Monte Carlo computer program which can be used to calculate the transmission probabilities for two cylindrical tubes in series is described, as well as the technique of adding directed mass motion to the random thermal motion of the molecules. The results for a simple straight cylindrical duct are compared to other solutions, and angle-of-attack effects are examined. Complex systems for which no adequate solutions have previously been found are easily analysed by this method. The Fortran instructions are listed along with a typical program solution.

*Quena*

I. INTRODUCTION

While the properties of rarefied gas flow have been studied in detail for many years, there still exists, in many respects, a real lack of understanding of the physics of the problem. Gross properties such as the transmission probability (the probability that, if a molecule enters a system at point 1, it will exit the system at point 2), the spatial distribution at, near, or in the system, etc., have been determined for very simple systems (i.e., cylindrical ducts, rectangular ducts, infinitely wide parallel plates, etc.) when the density is sufficiently low that the system is in the free molecular flow regime. However, the condition for free molecular flow is not exact since all the parameters which affect the flow properties are not known. The Knudsen number is normally defined as the ratio of the mean free path ( $\lambda$ ) to the characteristic dimension (L) of the system. When the Knudsen number is large ( $Kn = 10$  or higher), the system is said to be in free molecular flow. But if one is considering a cylindrical duct as a system, it is not known if the characteristic length is the diameter of the duct, the length of the duct, or some dimension resulting from a combination of these values. For very large mean free paths and relatively small length-to-diameter ratios for the duct, the exact Knudsen number is not important. When the mean free path of the molecule is near the characteristic length ( $10 \geq K \geq .1$ ), the flow regime is called transition flow, for which no sufficient description is presently available.

These problems are greatly magnified when one considers complex systems such as orifice-restricted tubes, baffles in tubes, elbows, etc., and flow where there is relative motion between the system and the gas. There is a wealth of information in the literature about this problem with most of the solutions anchored on the excellent work of Clausing [1]. His approach and techniques have been followed with some improvements in accuracy. It was not until Davis in 1959 [2] that any significantly different approach to the problem was developed. In a series of extremely useful papers [2,3,4,5], Davis and his co-workers made a significant contribution to the field of vacuum system design. Because of the obvious success of Davis' approach and the belief that this approach could be used for solutions of many problems, the writer has developed programs very similar to those of Davis to look at several types of problems. Some results of these programs have been published [6,7]. These programs consider complex geometrical systems, relative mass motion, reflection coefficients, spatial distributions, etc. Since many requests for information about these programs have been received, it was decided to present in detail the program for a simple system. This paper describes the Fortran computer program for the Monte Carlo calculations of the transmission probability for two co-axial cylinders connected in series with provisions for relative mass motion directed along the axis of the system or at any arbitrary angle with respect to the axis. Only the basic program will be described in detail. Modifications which have been incorporated to determine flux distributions, spatial distributions, capture coefficients, etc., will be only briefly mentioned since these aspects can be added to the basic program as the user desires. The program has not been refined. It was not the intent of this study to develop an optimum, time-saving computer solution. A skilled programmer could easily reduce the computing time. However, it was felt that the 8 minutes it takes for the present computer solution on a CDC-3200 (or 2 to 4 minutes on an IBM-7094) was reasonable. Most of the effort has been spent on the physics of the problem.

## II. DESCRIPTION OF THE PROGRAM

### A. ASSUMPTIONS

The following basic assumptions of the flow parameters of the molecules are consistent with most approaches:

1. The molecules enter the cylindrical duct with a direction proportional to the cosine of the angle between the normal to the entrance plane and the direction of the molecule. If the molecule strikes a wall, it leaves the wall in a direction proportional to the cosine of the angle between the normal to wall at the point of collision and the direction of the molecule.

2. The mean free path is sufficiently large such that collisions between molecules can be ignored. (Modifications to this basic program are not limited by this assumption; however, no description of these modifications will be made in this paper).

## B. MODEL DESCRIPTION

The model for this study is shown in Figure 1, where we see the two cylindrical tubes in series with a common axis. For convenience, the radius of tube 1 is given the value of one. The configuration to be studied is then described in input parameters for the following parameters. (The symbols enclosed by parentheses refer to the parameters in the Fortran program, Figure 3):

1. The length of tube 1, (Q).
2. The length of tube 2, (QL2).
3. The radius of the entrance orifice to tube 1, (A).
4. The radius of the exit orifice of tube 1, (B).
5. The diameter of tube 2, (CL2).
6. The radius of the exit orifice of tube 2, (DL2).

In Figure 1, tube 2 is shown to be larger in diameter than tube 1. The program is not restricted to this configuration and can be used with tube 2 having any diameter. If QL2 is zero, tube 2 is not considered by the program.

## C. THE MONTE CARLO PROCEDURE

In the Monte Carlo procedures, molecular histories are generated for a large number of molecular paths through the system. The properties of each molecule are recorded depending on the parameters of interest. This paper will consider only the tabulations of the number of molecules which exit the system at either exit plane as a function of the number of collisions with the walls (both the tube walls and the orifice plates).

A flow diagram for the computer program is shown in Figure 2; the Fortran print-out for the program is shown in Figure 3. The history for a molecule begins with instruction 2010. (The program instructions preceding number 2010 merely performs bookkeeping functions, presets counters, reads the input parameters, etc.) The initial coordinates of the molecule on the entrance plane are given by (with the axis of the cylinders being the positive X axis)



$$Y_1 = 1 - 2R_1$$

$$Z_1 = 1 - 2R_2,$$

where  $R_1$  and  $R_2$  are random numbers. (There are numerous methods to calculate pseudo-random numbers in the literature.) These coordinates can fall anywhere within the square bounded by  $y = \pm A$ , and  $z = \pm A$ . The next instruction selects only those points within the circle  $Y_1^2 + Z_1^2 = A^2$ . Instructions 2011 through 2142 calculate a cosine-bias angle.

If mass motion is to be considered, the instructions from 2555 through 2141 are used to calculate the random thermal speed of the molecule. This is explained more fully in the next section. Direction cosines for the molecular trajectory are determined from 2556 through 2020. Instruction 2024 tests to see if the molecule hits the exit plane of the tube. If it does not hit this plane, it must hit the wall of tube between  $X = 0$  and  $X = L(Q)$ . The point of collision is calculated by instructions 3026 through 3035. At this point, the molecule is reflected diffusely from the tube wall by the instructions 3039 up to 3045. The instruction before 3045 senses  $U_1$  to see which direction the molecule will go. Instructions 3045 to 3052 and 3060 to 3063 tests to see if the molecule hits the end planes of the cylinder. If it does not, the  $x$  coordinate of the point of collision with the tube wall is calculated by 3068 to 3642. The remainder of the program repeats these tests for molecules in tube 2.

#### D. SPEED DISTRIBUTION

Although the procedure for combining the relative mass motion to the random thermal motion is straightforward, it needs some explanation. In the program, the position of a molecule at the entrance plane of the system is determined, and the direction cosines for its motion considering only the cosine bias are calculated. For no mass motion, the molecules whose direction cosine  $U_1$ , because of its thermal velocity, is negative will not enter the system. This is not necessarily so when there is mass motion. The magnitude of the thermal speed of the molecule is chosen by the instructions starting at 2555 in the program. Figure 5 presents a histogram of the thermal speed as selected by this procedure. From this distribution, one can determine the most probable speed to be 1.414, the root-mean-square speed to be 1.777, and the average or mean speed to be 1.596. Also shown in Figure 5 is the theoretical Maxwellian-Boltzman distribution. One can see that this calculation satisfactorily approximates the Maxwellian distribution.

Using the direction cosines, one can then determine the magnitude of each component of the speed. Mass motion can then be added to the thermal

speed and new direction cosines for the actual path of the molecule relative to the moving tube. One small consideration must be made, however. This consideration is concerned with the relationship of the thermal speed of the molecules entering the system to the thermal speed of the molecules in the reservoir. From the kinetic theory of gases, one knows that the number of molecules striking the small area is given by

$$\frac{1}{4} V dN_V,$$

where  $V$  is the speed of the molecule and

$$dN_V = \frac{4N}{\sqrt{\pi}} \beta^3 V^2 e^{-\beta^2 V^2} dV$$

$$\beta = \frac{m}{2kT}.$$

The most probable speed of the molecules is found by setting the first derivative of function above equal to zero.

$$\frac{d}{dV} \left[ \frac{1}{4} V \frac{4N}{\sqrt{\pi}} \beta^3 V^2 e^{-\beta^2 V^2} \right] = 0$$

From this we find that the most probable speed of the molecules leaving the reservoir is given by

$$\left( V_{\text{exiting}} \right)_{\text{mp}} = \sqrt{3/2} \left( V_{\text{reservoir}} \right)_{\text{mp}}.$$

The distribution shown in Figure 5 is for the molecules exiting the reservoir (or entering the tube). Since the speed ratio is defined as the ratio of the mass motion to the most probable motion of the gas molecules in the reservoir, the mass motion vector which is added to the thermal speed vector must be expressed in terms of the reservoir speed. Thus, the mass motion speed vector is given by

$$\frac{1.414 S}{\sqrt{3/2}} = 1.155 S,$$

where  $S$  = speed ratio.

Figure 6 shows a histogram for the speed distribution with mass motion. The solid curve represents the Monte Carlo solution for the general expression of the velocity distribution

$$\frac{d^3N_V}{N} = \left(\frac{m}{2\pi kT}\right)^{3/2} e^{-\frac{m}{2kT} [(V_x - SI)^2 + V_y^2 + V_z^2]} dV_x dV_y dV_z$$

where  $N = 10,000$  and  $SI = 8 \left(v_{\text{reservoir}}\right)_{\text{mp}}$ .

The root mean square speed was found to be 9.384 and  $\bar{V}$ , 9.330. The dashed curve shows the distribution of molecules that actually enter the tube for the same mass motion. It is seen that, although the rms speed and the mean speed are approximately equal, the actual distributions are very different. This difference is a result of the cosine distribution of the entering molecules. In Figure 6, the peak on the program distribution at  $V = 8.5$  results from the addition of the mass motion to those molecules at the entrance plane which were traveling opposite to the direction of mass motion, while the peak at  $V = 10.0$  results from the addition of mass motion with the molecules which were traveling in the same direction as the mass motion.

### III. RESULTS

The transmission probabilities for cylindrical ducts at various speed ratios have been calculated by Pond [8], by deLeeuw and Rothe [9], and by Hughes [10]. While Pond presents his data directly as transmission probabilities, it is quite difficult to accurately obtain values from his figures. Rothe and deLeeuw present their data as a pressure ratio; some simple calculations are required to convert the data into transmission probabilities. The results of this operation are presented in Table 1 and in Figures 7 through 9. Figures 10 through 15 present data from the Monte Carlo solutions with the solid line representing the values interpolated from the UTIA data. It is quite noticeable that the Monte Carlo data for all  $L/A$  values are slightly lower than the UTIA data below speed ratio of 5. For the zero speed ratio data, the Monte Carlo values are consistently within  $\pm 5$  per cent of other solutions. For speed ratios up to 4, the difference between the Monte Carlo data and the UTIA data is between 5 and 10 per cent; however, the Monte Carlo data are always the smaller. No explanation of this can be given at this time.

All the previous results are for the case where the mass motion is directed along the axis of the tube. In general, the mass motion will be at any angle with respect to the axis of the tube. Defining the angle of attack to be the angle between the direction of mass motion and the axis of the tube, Figure 16 presents the Monte Carlo data for tubes of  $L/A = 2$ , and speed ratios of 2, at various angles of attack. It is interesting to note that, at an angle of attack of 90 degrees, the transmission probability (0.35) is not the same as the transmission probability for zero speed ratio (0.51).

In all the data presented so far, only a single tube has been considered. Normally, a measuring system in free molecule flow consists of a gauge volume connected to the free stream by an orifice-restricted tube or a series of tubes. The techniques for calculating the transmission probability for composite systems with zero mass motion have been examined by several investigators [7,11,12]. To see if these same approaches could be used for composite systems with mass motions, Monte Carlo calculations were made for an orifice-restricted tube of  $L/A = 8$ ,  $A_1/A_2 = .5$ , and for various speed ratios. So that any differences which might appear would be the result of the procedure, the Monte Carlo transmission probability values for a tube of  $L/A = 4$  were used in the calculations. Figure 17 presents the results of these calculations. Deviations up to approximately 15 per cent are observed between the calculated values and the Monte Carlo values. Tube lengths of  $L/A = 2$  and 4 were also examined (although not presented in this report). Deviations from the theoretical values for these tubes were quite small, i.e., 2 to 4 per cent. The standard techniques for coupling orifices and tubes in free molecular flow may be used for mass motion studies. The deviation from these values increased with increasing tube length, but decreased with increasing mass motion.

For the case of two tubes with transmission probabilities  $\alpha_1$  and  $\alpha_2$  corresponding to tube  $L_1/A_1$  and  $L_2/A_2$ , respectively, Reference 7 shows that the total transmission probability for zero mass motion is given by

$$\frac{1}{\alpha_T} = \frac{1}{\alpha_1} + \frac{A_1^2}{A_2^2 \alpha_2} - \frac{A_1^2}{A_2^2} .$$

Assuming this equation to be valid for the case where there is mass motion such that  $\alpha(L/A)$  becomes  $\alpha(L/A,S)$ , calculations were made for two values of  $L/A$  and various speed ratios and are compared with Monte Carlo data for the same parameters in Figure 19. There is excellent agreement between the calculated values and the Monte Carlo values.

Figure 20 presents a comparison of Monte Carlo solutions for transmission probabilities for cylindrical ducts and angles of attack with data from Reference 10 for the same configuration. Again, there is excellent agreement between the two methods.

#### IV. CONCLUSIONS

A simple Monte Carlo computer program similar to the one developed by Davis has been modified to permit investigation of the influence of mass motion with random orientation with respect to the tube. Comparison of results from this program to numerical solutions of other investigations show the agreement of the different methods to be excellent. Using the inherent advantages of this program, such as orifice studies, series of tubes, etc., the general coupling equations developed for zero mass flow appear to be applicable to flows with relative mass motion.

After this comparison of numerical methods to Monte Carlo methods for simple tubes, a great amount of confidence may be placed in attempting investigations of parameters such as reflection coefficients, thermal and accommodation coefficients, etc. Also rather simple modifications to this program allow one to examine ducts of other types, i.e., elliptical and conical.

TABLE I

TRANSMISSION PROBABILITIES FOR CYLINDRICAL DUCTS  
AT VARIOUS SPEED RATIOS

(CALCULATED FROM UNIVERSITY OF TORONTO REPORT NO. 88,  
DECEMBER 1962, BY J. H. DELEEUEW AND D. E. ROTHE)

| L/A       | K(S,L/A) |         |         |         |
|-----------|----------|---------|---------|---------|
|           | S=0      | S=0.02  | S=0.05  | S=0.1   |
| .010021   | .995015  | .995073 | .995179 | .995317 |
| .013338   | .993375  | .993433 | .993630 | .993760 |
| .017752   | .991202  | .991355 | .991456 | .991753 |
| .023628   | .988324  | .988477 | .988668 | .989040 |
| .031449   | .984520  | .984767 | .985043 | .985482 |
| .041859   | .979501  | .979747 | .980111 | .980788 |
| .055714   | .972901  | .973239 | .973775 | .974507 |
| .074155   | .964256  | .964684 | .965299 | .966416 |
| .098701   | .952988  | .953595 | .954368 | .955765 |
| .131371   | .938402  | .939091 | .940191 | .941927 |
| .174854   | .919688  | .920629 | .921947 | .924150 |
| .232731   | .895950  | .897040 | .898725 | .901429 |
| .309765   | .866283  | .867671 | .869681 | .872967 |
| .412298   | .829888  | .831538 | .833904 | .837829 |
| .548769   | .786257  | .788124 | .790854 | .795436 |
| .730410   | .735367  | .737397 | .740476 | .745562 |
| .972176   | .677841  | .679974 | .683233 | .688666 |
| 1.293971  | .614986  | .617159 | .620498 | .626050 |
| 1.722267  | .548766  | .550917 | .554137 | .559610 |
| 2.292342  | .481741  | .483769 | .486810 | .491991 |
| 3.051106  | .416713  | .418547 | .421327 | .426000 |
| 4.061021  | .355790  | .357391 | .359827 | .363959 |
| 5.405230  | .299834  | .301212 | .303291 | .306845 |
| 7.194348  | .249045  | .250213 | .251962 | .254952 |
| 9.575701  | .203677  | .204633 | .206081 | .208559 |
| 12.745267 | .164051  | .164837 | .166018 | .168025 |
| 16.963960 | .130320  | .130944 | .131882 | .133499 |
| 22.578970 | .102309  | .102799 | .103545 | .104813 |
| 30.052592 | .079535  | .079916 | .080503 | .081495 |
| 40.000000 | .061367  | .061667 | .062114 | .062879 |

TABLE I CONTINUED

TRANSMISSION PROBABILITIES FOR CYLINDRICAL DUCTS  
AT VARIOUS SPEED RATIOS

(CALCULATED FROM UNIVERSITY OF TORONTO REPORT NO. 88,  
DECEMBER 1962, BY J. H. DELEEUEW AND D. E. ROTHE)

| L/A       | K(S,L/A) |         |         |         |
|-----------|----------|---------|---------|---------|
|           | S=0.2    | S=0.3   | S=0.5   | S=0.75  |
| .010021   | .995636  | .995930 | .996414 | .997009 |
| .013338   | .994209  | .994534 | .995239 | .995994 |
| .017752   | .992247  | .992786 | .993668 | .994687 |
| .023628   | .989720  | .990392 | .991619 | .992945 |
| .031449   | .986405  | .987248 | .988868 | .990612 |
| .041859   | .981939  | .983122 | .985209 | .987524 |
| .055714   | .976090  | .977638 | .980400 | .983435 |
| .074155   | .968454  | .970379 | .974047 | .978019 |
| .098701   | .958367  | .960922 | .965667 | .970845 |
| .131371   | .945314  | .948589 | .954683 | .961429 |
| .174854   | .928441  | .932566 | .940359 | .949078 |
| .232731   | .906855  | .912034 | .921860 | .932991 |
| .309765   | .879561  | .886005 | .898220 | .912209 |
| .412298   | .845703  | .853442 | .868328 | .885554 |
| .548769   | .804512  | .813570 | .831181 | .851933 |
| .730410   | .755763  | .765950 | .786133 | .810390 |
| .972176   | .699656  | .710758 | .732956 | .760292 |
| 1.293971  | .637380  | .648912 | .672337 | .701856 |
| 1.722267  | .570796  | .582290 | .605929 | .636333 |
| 2.292342  | .502600  | .513549 | .536385 | .566302 |
| 3.051106  | .435683  | .445744 | .466882 | .494972 |
| 4.061021  | .372523  | .381433 | .400314 | .425685 |
| 5.405230  | .314215  | .321907 | .338272 | .360475 |
| 7.194348  | .261150  | .267640 | .281523 | .300456 |
| 9.575701  | .213679  | .219073 | .230612 | .246424 |
| 12.745267 | .172202  | .176594 | .186016 | .198984 |
| 16.963960 | .136841  | .140364 | .147934 | .158386 |
| 22.578970 | .107458  | .110245 | .116238 | .124537 |
| 30.052592 | .083566  | .085744 | .090446 | .096955 |
| 40.000000 | .064482  | .066169 | .069811 | .074866 |

TABLE I CONTINUED

TRANSMISSION PROBABILITIES FOR CYLINDRICAL DUCTS  
AT VARIOUS SPEED RATIOS

(CALCULATED FROM UNIVERSITY OF TORONTO REPORT NO. 88,  
DECEMBER 1962, BY J. H. DELEEUEW AND D. E. ROTHE)

| L/A       | K(S,L/A) |         |         |         |
|-----------|----------|---------|---------|---------|
|           | S=1.0    | S=1.5   | S=2.0   | S=3.0   |
| .010021   | .997460  | .998153 | .998587 | .999066 |
| .013338   | .996636  | .997533 | .998118 | .998727 |
| .017752   | .995520  | .996726 | .997500 | .998314 |
| .023628   | .994043  | .995648 | .996680 | .997738 |
| .031449   | .992060  | .994198 | .995579 | .997046 |
| .041859   | .989455  | .992289 | .994109 | .996107 |
| .055714   | .985974  | .989744 | .992171 | .994793 |
| .074155   | .981379  | .986356 | .989582 | .993026 |
| .098701   | .975260  | .981871 | .986122 | .990741 |
| .131371   | .967202  | .975905 | .981549 | .987666 |
| .174854   | .956595  | .967981 | .974281 | .983536 |
| .232731   | .942654  | .957496 | .967353 | .978117 |
| .309765   | .924480  | .943676 | .956601 | .970900 |
| .412298   | .900963  | .925521 | .977487 | .961324 |
| .548769   | .870840  | .901780 | .923601 | .948489 |
| .730410   | .832991  | .871117 | .898954 | .931629 |
| .972176   | .786407  | .832080 | .866865 | .909231 |
| 1.293971  | .730797  | .783488 | .825723 | .879799 |
| 1.722267  | .666936  | .724985 | .774219 | .841413 |
| 2.292342  | .597051  | .657693 | .712129 | .792279 |
| 3.051106  | .524404  | .584387 | .641035 | .731374 |
| 4.061021  | .452652  | .509051 | .564575 | .659777 |
| 5.405230  | .384325  | .435216 | .486945 | .581021 |
| 7.194348  | .320979  | .365417 | .411695 | .499720 |
| 9.575701  | .263678  | .301455 | .341523 | .420331 |
| 12.745267 | .213191  | .244575 | .278317 | .346376 |
| 16.963960 | .169876  | .195418 | .223168 | .280303 |
| 22.578970 | .133684  | .154127 | .176499 | .223239 |
| 30.052592 | .104149  | .124756 | .138040 | .175498 |
| 40.000000 | .080462  | .093050 | .106967 | .136563 |



TABLE I CONTINUED

TRANSMISSION PROBABILITIES FOR CYLINDRICAL DUCTS  
AT VARIOUS SPEED RATIOS

(CALCULATED FROM UNIVERSITY OF TORONTO REPORT NO. 88,  
DECEMBER 1962, BY J. H. DELEEuw AND D. E. ROTHE)

| L/A       | K(S,L/A) |         |         |         |
|-----------|----------|---------|---------|---------|
|           | S=4.0    | S=5.0   | S=10    | S=20    |
| .010021   | .999324  | .999422 | .999728 | .999854 |
| .013338   | .999078  | .999232 | .999621 | .999817 |
| .017752   | .998780  | .999003 | .999504 | .999755 |
| .023628   | .998319  | .998668 | .999334 | .999668 |
| .031449   | .997809  | .998212 | .999126 | .999556 |
| .041859   | .997074  | .997655 | .998812 | .999407 |
| .055714   | .996051  | .996861 | .998422 | .999218 |
| .074155   | .994749  | .995837 | .997901 | .998948 |
| .098701   | .993071  | .994415 | .997208 | .998606 |
| .131371   | .990710  | .992590 | .996293 | .998146 |
| .174854   | .987687  | .990126 | .995078 | .997530 |
| .232731   | .983614  | .986862 | .993430 | .996716 |
| .309765   | .978170  | .982532 | .991253 | .995628 |
| .412298   | .970961  | .976744 | .988376 | .994182 |
| .548769   | .961334  | .969042 | .984521 | .992262 |
| .730410   | .948588  | .958846 | .979401 | .989701 |
| .972176   | .931699  | .945256 | .972597 | .986288 |
| 1.293971  | .909277  | .927241 | .963531 | .981747 |
| 1.722267  | .879753  | .903438 | .951487 | .978051 |
| 2.292342  | .841134  | .872023 | .935498 | .967719 |
| 3.051106  | .791366  | .830957 | .914288 | .956995 |
| 4.061021  | .729290  | .778163 | .886246 | .942842 |
| 5.405230  | .655847  | .712652 | .849368 | .923969 |
| 7.194348  | .574787  | .635915 | .801390 | .899044 |
| 9.575701  | .491180  | .552293 | .740265 | .866123 |
| 12.745267 | .410023  | .467380 | .665846 | .822983 |
| 16.963960 | .335220  | .386354 | .581363 | .767235 |
| 22.578970 | .269265  | .313020 | .493029 | .697465 |
| 30.052592 | .213006  | .249408 | .407378 | .615228 |
| 40.000000 | .166509  | .195940 | .329503 | .526056 |

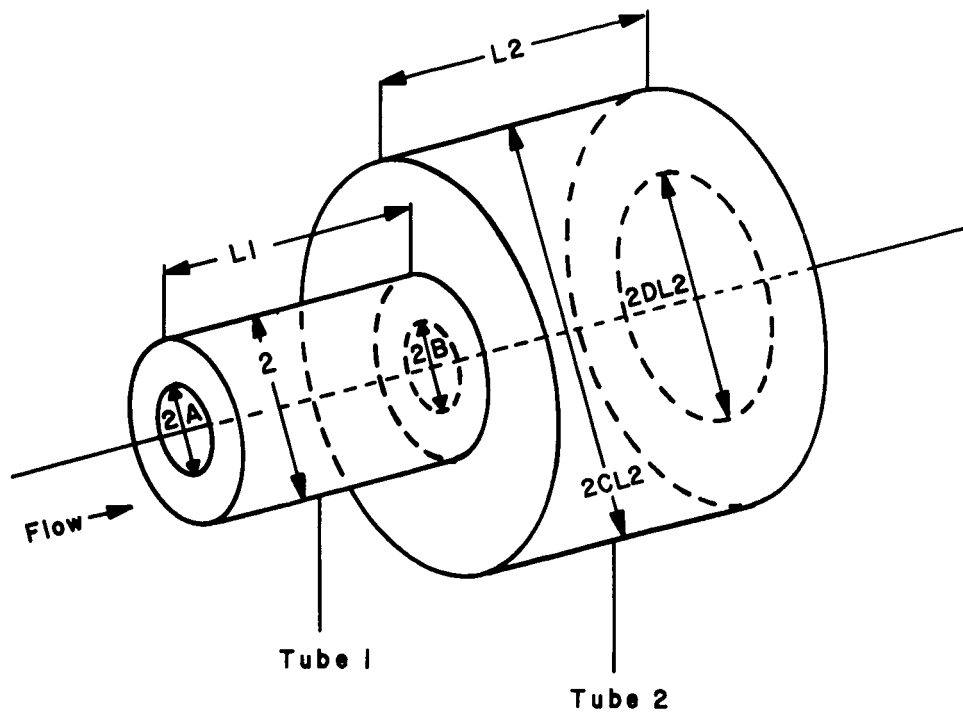


FIG. 1. MODEL FOR MONTE CARLO PROGRAM FOR  
TRANSMISSION PROBABILITY CALCULATION

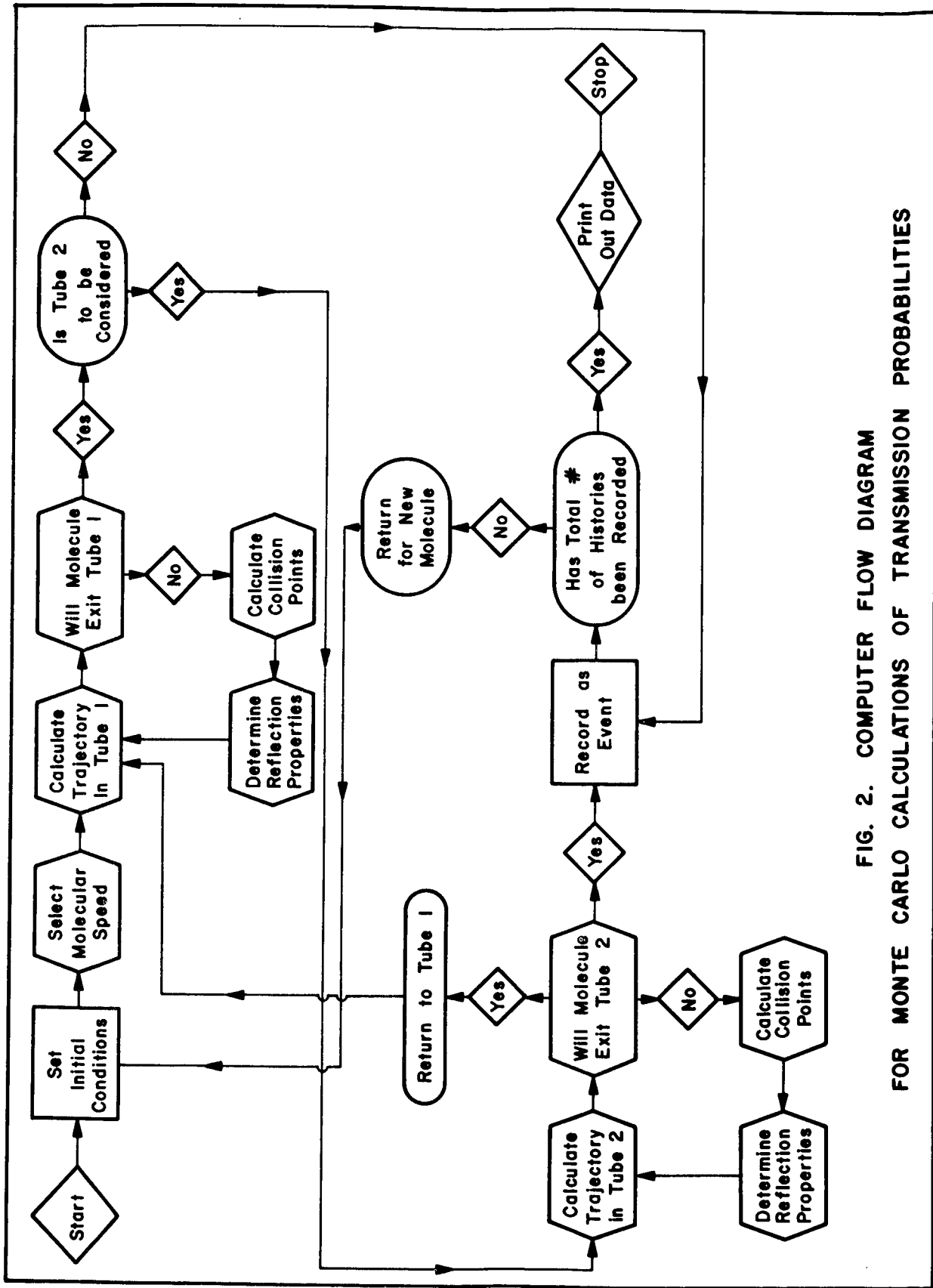


FIG. 2. COMPUTER FLOW DIAGRAM FOR MONTE CARLO CALCULATIONS OF TRANSMISSION PROBABILITIES

```

C   TRANSMISSION PROBABILITIES FOR CYLINDRICAL DUCTS
COMMON ISTART(2)
DIMENSION COUNT(3,250)
1  FORMAT(F9.4,5XF9.4,5XF9.4,5XF5.0)
2  FORMAT(F9.4,5XF9.4,5XF9.4,5XF9.4)
3  FORMAT(1X9HCOLLISION 5X9HEXIT AT L7X9HEXIT AT O7X10HEXIT AT L2)
4  FORMAT (2XI3, 12XF5.0, 12XF5.0, 12XF5.0)
5  FORMAT(11X3HX=L2XF5.0,8X3HX=O2XF5.0,8X4HX=L22XF5.0,7X3HOUT2XF5.0)
6  FORMAT (F14.7)
7  FORMAT (2H27,2X1HT2XF14.7,5X1HV2XF14.7,5X1HW2XF14.7)
8  FORMAT (2H70,2X1HT2XF14.7,5X1HV2XF14.7,5X1HW2XF14.7)
9  FORMAT (1H )
37 FORMAT (5HX(IC), 2XF14.7, 5X8HCOLL NUM, 2XI3)
151 FORMAT (F5.3)
152 FORMAT(3HL1=F9.4,5X3HL2=F9.4,5X2HS=F9.4,5X5HTOTAL2XF5.0)
153 FORMAT(1X2HA=F9.4,6X2HP=F9.4,5X2HC=F9.4,5X2HD=F9.4)
260  FORMAT(1X44HMONTE CARLO SOLUTIONS OF FREE MOLECULAR FLOW)
261  FORMAT (1X25HTHROUGH CYLINDRICAL DUCTS)
267  FORMAT(1X16HAND SPEED RATIOS)
268  FORMAT(1X15HNO SPEED RATIOS)
270  FORMAT (1H1)
271  FORMAT(I4)
272  FORMAT (1X25HSPEED RATIO = K(1,154700))
604  FORMAT (6HX(IC1), F14.7,6HX(IC2),F14.7,8HCOLL NUM,I3)
605  FORMAT(1HYF14.7,1HZF14.7,1HXF14.7,2HD1F14.7,2HD2F14.7)
606  FORMAT(2HU1F14.7,2HU2F14.7,2HU3F14.7,2HAAE14.7,2HAEF14.7,F14.7)
611  FORMAT (5HX(IC), 2XF14.7, 7HCOLL NO, 2XI3)
650  FORMAT (O8, 5XO8)
8700  FORMAT (F9.4)
8701  FORMAT(1X26HANGLE OF ATTACK IN DEGREES, F9.4)
      PI = 3.1415926536
150  XL2T = 0.
7800  FORMAT (5X6HISTART, 5XO8, 5XO8)
      XOT = 0.
      YLT = 0.
      OUT = 0.
      ATST = 0.
      READ 1, O, QL2, SI, TMAX
      READ 2, A, B, CL2, DL2
      READ 650, ISTART(1), ISTART(2)
      READ 271, NCOLL
      READ 8700 , ANGLE
      PRINT 270
      PRINT 260
      PRINT 261
      IF (SI) 265,266,265

```

FIGURE 3

FORTRAN PROGRAM FOR MONTE CARLO CALCULATIONS OF  
FREE MOLECULAR FLOW THROUGH CYLINDRICAL DUCTS

```

265 PRINT 267
    PRINT 272
    GO TO 269
266 PRINT 268
269 PRINT 9
    PRINT 9
    PRINT 152, Q, QL2, SI, TMAX
    PRINT 153, A, B, CL2, DL2
    PRINT 8701, ANGLE
    PRINT 9
    DO 500 J = 1,3
    DO 500 K = 1,NCOLL
500 COUNT(J,K) = 0.
    AATAC = ANGLE*.0174532925
2010 Y1 = (1. - 2.*FLAT(ISTART))*A
    Z1 = (1. - 2.*FLAT(ISTART))*A
    D = Y1*Y1 + Z1*Z1
    ICOLL = 1
    IF (D - A*A ) 2011,2010,2010
2011 R1 = FLAT(ISTART)
    R2 = FLAT(ISTART)
    IF(R1 - R2) 2013,2011,2012
2012 CTHET = R1
    GO TO 2142
2013 CTHET = R2
2142 STHET = SQRT(1. - CTHET**2)
    SP = 1.
    IF (SI) 2555,2556,2555
2555 XT = 0.
    YT = 0.
    ZT = 0.
    DO2557 J = 1,12
    XA = FLAT(ISTART)
    YA = .FLAT(ISTART)
2558 YT = YT + YA
    ZA = FLAT(ISTART)
2559 ZT = ZT + ZA
2557 XT = XT + XA
    VX = XT - 6.
    VY = YT - 6.
    VZ = ZT - 6.
    SP = SQRT(VX*VX + VY*VY + VZ*VZ)
    IF (FLAT(ISTART) - .5) 2141,2011,2556
2141 CTHET = -CTHET
2556 PSI = FLAT(ISTART)*2.*PI
    AA = CTHET* SP + SI*1.154700*COSE(AATAC)
    AB = STHET*COSE(PSI)*SP + SI*1.154700*SINE(AATAC)
    AC = STHET * SINE(PSI)*SP
    DENO = SQRT(AA*AA + AB*AB + AC*AC)
    U1 = AA/DENO
    IF (U1 - 0.) 2011,2011,2998
2998 U2 = AB/DENO

```

FIGURE 3 CONTINUED

```

      U3 = AC/DENO
      ATST = ATST + 1.
2021 DELTA = U2/U1
      DELTB = U3/U1
2020 X1 = 0.
      XIXL = 0
      XIXO = 0.
      YIXL = DELTA*Q + Y1
      ZIXL = DELTB*Q + Z1
2024 IF (YIXL*YIXL + ZIXL*ZIXL - 1.) 4100,4100,3026
3026 T = DELTA*Y1 + DELTB*Z1
      V = DELTA*DELTA + DELTB*DELTB
      W = Y1*Y1 + Z1*Z1 - 1.
3029 P = SQRT(T*T - V*W)
      XIC = (-T+P)/V + X1
3034 IF (XIC - Q) 3035,3035,3036
3036 PRINT 37, XIC , ICOLL
      GO TO 7115
3035 YIC = DELTA*(XIC - X1) + Y1
      ZIC = DELTB*(XIC - X1) + Z1
3039 STAU = ZIC
      CTAU = YIC
3025 ICOLL = ICOLL + 1
      IF (ICOLL - NCOLL) 3341,7115,7115
3341 IF (XIC ) 3036,3036,3040
3040 R1 = FLAT(ISTART)
      R2 = FLAT(ISTART)
      IF (R1 - R2) 3042,3040,3041
3041 CTHET = R1
      GO TO 3043
3042 CTHET = R2
3043 STHET = SQRT(1. - CTHET**2)
3044 PSI = FLAT(ISTART)*2.*PI
      U1 = SINE(PSI)*STHET
      U2 = -CTHET*CTAU - STHET*COSE(PSI)*STAU
      U3 = -CTHET*STAU + STHET*COSE(PSI)*CTAU
      DELTA = U2/U1
      DELTB = U3/U1
      IF (U1 ) 3060,3045,3045
3045 YIXL = DELTA*(Q - XIC) + YIC
      ZIXL = DELTB*(Q - XIC) + ZIC
      X1 = XIC
      Y1 = YIC
      Z1 = ZIC
3052 IF (YIXL*YIXL + ZIXL*ZIXL - 1.) 4100,4100,3068
3060 YIXO = DELTA*(-XIC) + YIC
      ZIXO = DELTB*(-XIC) + ZIC
      XIXO = 0.
      X1 = XIC
      Y1 = YIC
      Z1 = ZIC
3063 IF (YIXO*YIXO + ZIXO*ZIXO - 1.) 5110,5110,3068

```

FIGURE 3 CONTINUED

```

3068 XIXO = 0.
3069 T = DELTA*Y1 + DELTR*Z1
      V = DELTA*DELTA + DELTR*DELTR
      IF (U1) 3691,3039,3692
3691 XIC = -ARCF((2.*T)/V) + X1
      GO TO 3034
3692 XIC =  ARCF((2.*T)/V) + X1
      GO TO 3034
4100 IF (YIXL*YIXL + ZIXL*ZIXL - B*B) 4101,4799,4799
4101 COUNT(1,ICOLL) = COUNT(1,ICOLL) + 1.
      XLT = XLT + 1.
      IF (QL2) 7120,7120,6170
4799 ICOLL = ICOLL + 1
      IF (ICOLL - NCOLL) 4800,7115,7115
4800 R1 = FLAT(ISTART)
      R2 = FLAT(ISTART)
      IF(R1 - R2) 4801,4800,4802
4801 CTHET = - R2
      GO TO 4803
4802 CTHET = - R1
4803 STHET = SORTF(1. - CTHET**2)
      PSI = FLAT(ISTART)*2.*PI
      U1 = CTHET
      U2 = STHET*COSE(PSI)
      U3 = STHET*SINE(PSI)
      DELTA = U2/U1
      DELTR = U3/U1
4804 YIXO = DELTA*(-XIXL) + YIXL
      ZIXO = DELTR*(-XIXL) + ZIXL
      X1 = XIXL
      Y1 = YIXL
      Z1 = ZIXL
      IF (YIXO*YIXO + ZIXO*ZIXO - 1.) 5110,5110,4805
4805 T = DELTA*YIXL + DELTR*ZIXL
      V = DELTA*DELTA + DELTR*DELTR
      W = YIXL*YIXL + ZIXL*ZIXL - 1.
      XIC = (-T-SORTF(T*T - V*W))/V + XIXL
      GO TO 3035
5110 IF (YIXO*YIXO + ZIXO*ZIXO - A*A) 5111,5810,5810
5111 COUNT (2,ICOLL) = COUNT(2,ICOLL) + 1.
      XOT = XOT + 1.
      GO TO 7120
5810 ICOLL = ICOLL + 1
      IF (ICOLL - NCOLL) 5811,7115,7115
5811 R1 = FLAT(ISTART)
      R2 = FLAT(ISTART)
      IF (R1 - R2) 5812,5811,5813
5812 CTHET = R2
      GO TO 5814
5813 CTHET = R1
5814 STHET = SORTF(1. - CTHET**2)
      PSI = FLAT(ISTART)*2.*PI

```

```

U1 = CTHET
U2 = STHET*COSE(PSI)
U3 = STHET*SINE(PSI)
Y1 = YIXO
Z1 = ZIXO
GO TO 2021
6170 XL2 = XIXL + OL2
    YXL2 = DELTA*(XL2 - XIXL) + YIXL
    ZXL2 = DELTR*(XL2 - XIXL) + ZIXL
    IF (YXL2*YXL2 + ZXL2*ZXL2 - CL2*CL2) 6170,6171,6171
6171 ICOLL = ICOLL + 1
    IF (ICOLL - NCOLL) 6172,7115,7115
6172 T = DELTA*YIXL + DELTR*ZIXL
    V = DELTA*DELTA + DELTR*DELTR
    W = YIXL*YIXL + ZIXL*ZIXL - CL2*CL2
    P = SORTF(T*T - V*W)
    XIC2 = ((-T+P)/V) + YIXL
    Y1 = YIXL
    Y1 = YIXL
    Z1 = ZIXL
6720 YIC2 = DELTA*(YIC2 - Y1) + Y1
    ZIC2 = DELTR*(YIC2 - Y1) + Z1
    STAU = ZIC2/CL2
    CTAU = YIC2/CL2
6173 R1 = FLAT(ISTART)
    R2 = FLAT(ISTART)
    IF (R1 - R2) 6175,6173,6174
6174 CTHET = R1
    GO TO 6176
6175 CTHET = R2
6176 STHET = SORTF(1. - CTHET**2)
    PSI = FLAT(ISTART)*2.*PI
    U1 = SINE(PSI)*STHET
    U2 = -CTHET*CTAU - STHET*COSE(PSI)*STAU
    U3 = -CTHET*STAU + STHET*COSE(PSI)*CTAU
    DELTA = U2/U1
    DELTR = U3/U1
    IF (U1) 6177,6173,6178
6177 YXON = DELTA*(XIXL - YIC2) + YIC2
    ZXON = DELTR*(XIXL - YIC2) + ZIC2
    IF (YXON*YXON + ZXON*ZXON - CL2*CL2) 6180,6180,6811
6178 YXL2 = DELTA*(XL2 - YIC2) + YIC2
    ZXL2 = DELTR*(XL2 - YIC2) + ZIC2
    IF (YXL2*YXL2 + ZXL2*ZXL2 - CL2*CL2) 6180,6811,6811
6180 IF (YXON*YXON + ZXON*ZXON - R*R) 6181,6181,6182
6181 YIXL = YXON
    ZIXL = ZXON
    GO TO 4804
6182 ICOLL = ICOLL + 1
    IF (ICOLL - NCOLL) 6183,7115,7115
6183 R1 = FLAT(ISTART)
    R2 = FLAT(ISTART)

```

FIGURE 3 CONTINUED



```

        IF (R1 - R2) 6184,6183,6185
6185 CTHET = R1
        GO TO 6186
6184 CTHET = R2
6186 STHET = SQRTF(1. - CTHET**2)
        PSI = FLAT(ISTART)*2.*PI
        U1 = CTHET
        U2 = STHET*COSE(PSI)
        U3 = STHET*SINE(PSI)
        DELTA = U2/U1
        DELTR = U3/U1
        YIXL = YXQN
        ZIXL = ZXQN
        GO TO 6170
6811 X1 = XIC2
        Y1 = YIC2
        Z1 = ZIC2
        ICOLL = ICOLL + 1
        IF (ICOLL - NCOLL) 6810,7115,7115
6810 T = DELTA*Y1 + DELTR*Z1
        V = DELTA*DELTA + DELTR*DELTR
        IF (U1) 6821,7115,6822
6821 XIC2 = -ARSE((2.*T)/V) + X1
        GO TO 6720
6822 XIC2 = ARSE((2.*T)/V) + X1
        GO TO 6720
6850 IF (YXL2*YYL2 + ZXL2*ZXL2 - DL2*DL2) 6860,6851,6851
6851 ICOLL = ICOLL + 1
        IF (ICOLL - NCOLL) 6852,7115,7115
6852 R1 = FLAT(ISTART)
        R2 = FLAT(ISTART)
        IF (R1 - R2) 6853,6852,6854
6853 CTHET = R2
        GO TO 6855
6854 CTHET = R1
6855 STHET = SQRTF(1. - CTHET**2)
        PSI = FLAT(ISTART)*2.*PI
        U1 = -CTHET
        U2 = STHET*COSE(PSI)
        U3 = STHET*SINE(PSI)
        DELTA = U2/U1
        DELTR = U3/U1
        YXQN = DELTA*(XIXL - X12) + YXL2
        ZXQN = DELTR*(XIXL - X12) + ZXL2
        IF (YXQN*YXQN + ZXQN*ZXQN - CL2*CL2) 6180,6180,6856
6856 T = DELTA*YXL2 + DELTR*ZXL2
        V = DELTA*DELTA + DELTR*DELTR
        W = YXL2*YXL2 + ZXL2*ZXL2 - CL2*CL2
        XIC2 = (-T-SQRTF(T**2 - V*W))/V + XL2
        Z1 = ZXL2
        Y1 = YXL2
        X1 = XL2

```

FIGURE 3 CONTINUED

```

        GO TO 6720
6860  COUNT (3,ICOLL) = COUNT(3,ICOLL) + 1.
        XL2T = XL2T + 1.
        GO TO 7120
7115  OUT = OUT + 1.
7120  IF (ATST - TMAX) 2010,7211,7211
7211  PRINT 3
        DO7140 J = 1,NCOLL
        K = J - 1
7140  PRINT 4, K, COUNT(1,J), COUNT(2,J), COUNT(3,J)
        PRINT 9
        PRINT 5, XLT, XOT, XL2T, OUT
        PRINT 9
        PRINT 7800, ISTART(1), ISTART(2)
9088  GO TO 150
        END

```

FIGURE 3 CONTINUED

MONTE CARLO SOLUTIONS OF FREE MOLECULAR FLOW  
 THROUGH CYLINDRICAL DUCTS  
 AND SPEED RATIOS  
 SPEED RATIO = K(1.154700)

1= 2.0000 L2= 0 S= .4640 TOTAL 9999  
 A= 1.0000 B= 1.0000 C= 0 D= 0  
 ANGLE OF ATTACK IN DEGREES 40.0000

| COLLISION | EXIT AT L | EXIT AT O | EXIT AT L2 |     |   |
|-----------|-----------|-----------|------------|-----|---|
| 0         | 2203      | 0         | 0          |     |   |
| 1         | 1209      | 2126      | 0          |     |   |
| 2         | 853       | 975       | 0          |     |   |
| 3         | 533       | 542       | 0          |     |   |
| 4         | 319       | 320       | 0          |     |   |
| 5         | 178       | 180       | 0          |     |   |
| 6         | 98        | 118       | 0          |     |   |
| 7         | 70        | 74        | 0          |     |   |
| 8         | 42        | 45        | 0          |     |   |
| 9         | 25        | 17        | 0          |     |   |
| 10        | 15        | 11        | 0          |     |   |
| 11        | 11        | 6         | 0          |     |   |
| 12        | 7         | 9         | 0          |     |   |
| 13        | 2         | 3         | 0          |     |   |
| 14        | 1         | 3         | 0          |     |   |
| 15        | 1         | 1         | 0          |     |   |
| 16        | 1         | 0         | 0          |     |   |
| 17        | 0         | 0         | 0          |     |   |
| 18        | 0         | 0         | 0          |     |   |
| 19        | 0         | 0         | 0          |     |   |
| 20        | 1         | 0         | 0          |     |   |
| 21        | 0         | 0         | 0          |     |   |
| 22        | 0         | 0         | 0          |     |   |
| 23        | 0         | 0         | 0          |     |   |
| 24        | 0         | 0         | 0          |     |   |
| 25        | 0         | 0         | 0          |     |   |
| 26        | 0         | 0         | 0          |     |   |
| 27        | 0         | 0         | 0          |     |   |
| 28        | 0         | 0         | 0          |     |   |
| 29        | 0         | 0         | 0          |     |   |
| 30        | 0         | 0         | 0          |     |   |
| 31        | 0         | 0         | 0          |     |   |
| 32        | 0         | 0         | 0          |     |   |
| 33        | 0         | 0         | 0          |     |   |
| 34        | 0         | 0         | 0          |     |   |
| 35        | 0         | 0         | 0          |     |   |
| 36        | 0         | 0         | 0          |     |   |
| 37        | 0         | 0         | 0          |     |   |
| 38        | 0         | 0         | 0          |     |   |
| 39        | 0         | 0         | 0          |     |   |
| 40        | 0         | 0         | 0          |     |   |
| 41        | 0         | 0         | 0          |     |   |
| 42        | 0         | 0         | 0          |     |   |
| 43        | 0         | 0         | 0          |     |   |
| 44        | 0         | 0         | 0          |     |   |
| 45        | 0         | 0         | 0          |     |   |
| 46        | 0         | 0         | 0          |     |   |
| 47        | 0         | 0         | 0          |     |   |
| 48        | 0         | 0         | 0          |     |   |
| 49        | 0         | 0         | 0          |     |   |
|           | X=L 5569  | X=O 4430  | X=L2 0     | OUT | 0 |

FIGURE 4 - TYPICAL COMPUTER RESULT

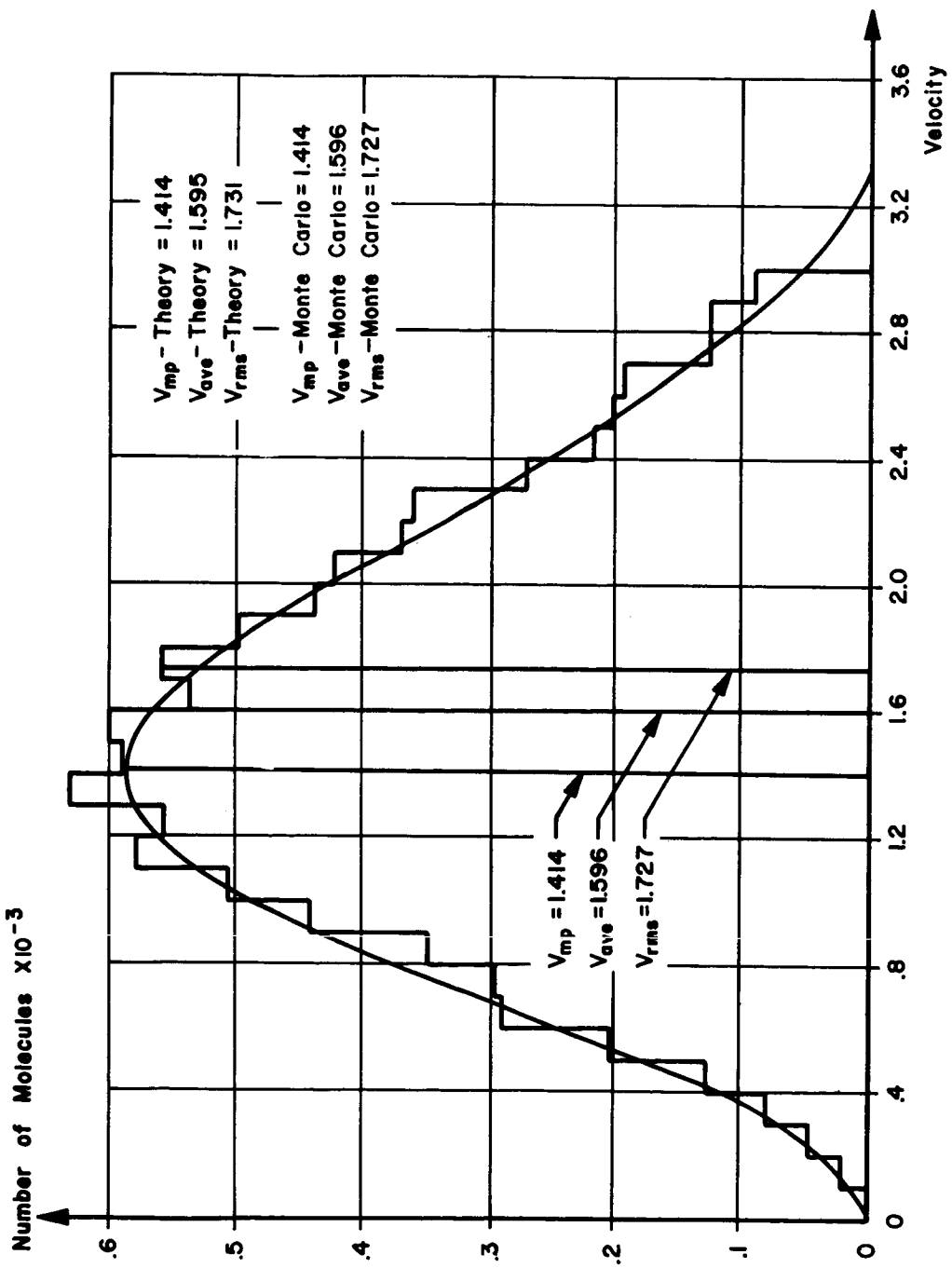


FIG. 5. DISTRIBUTION OF SPEEDS FROM MONTE CARLO CALCULATION

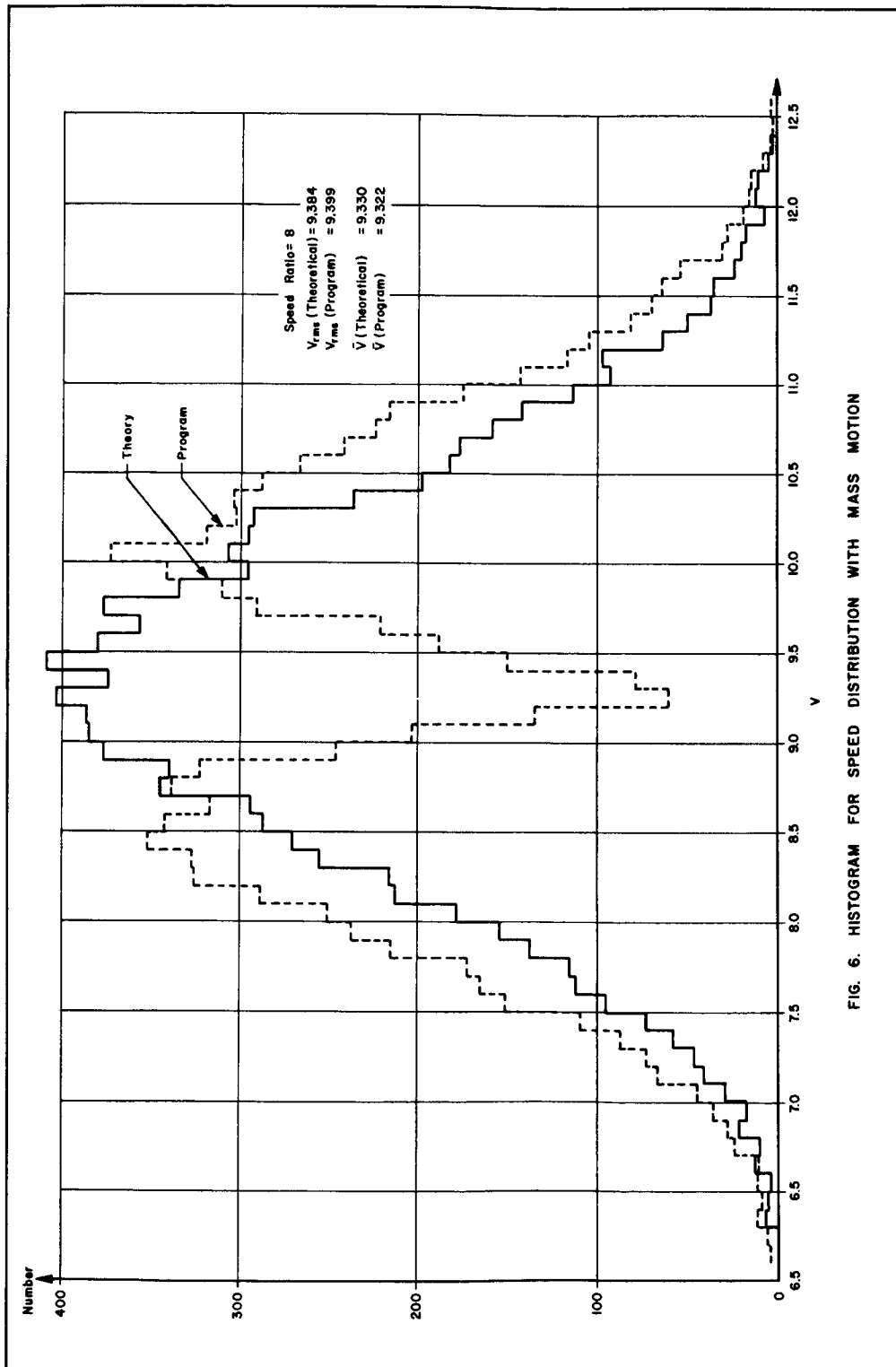


FIG. 6. HISTOGRAM FOR SPEED DISTRIBUTION WITH MASS MOTION

Calculated from UTIA Report # 88

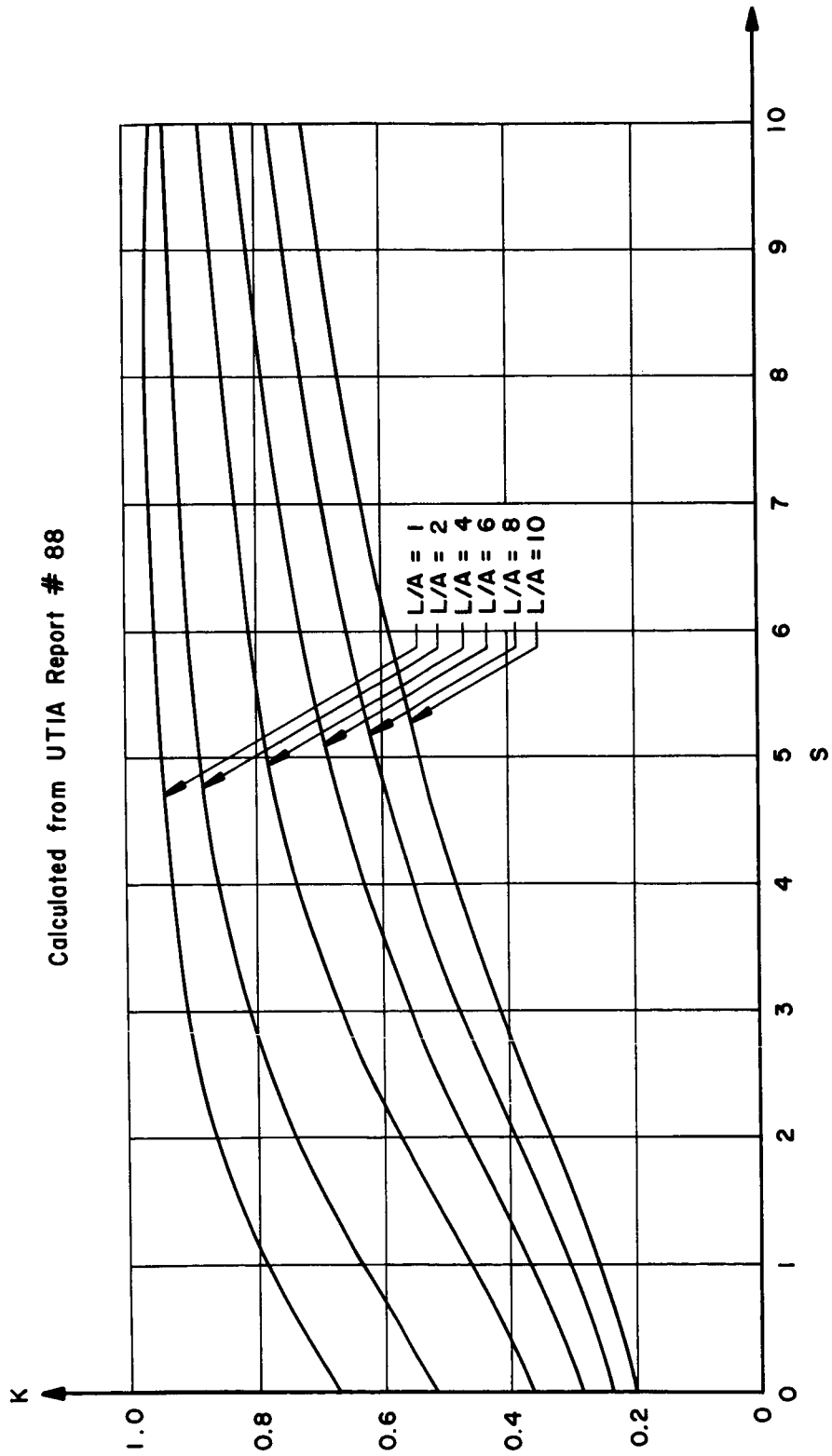


FIGURE 7. TRANSMISSION PROBABILITIES FOR CYLINDRICAL DUCTS AS A FUNCTION OF SPEED RATIO

Calculated from UTIA Report # 88

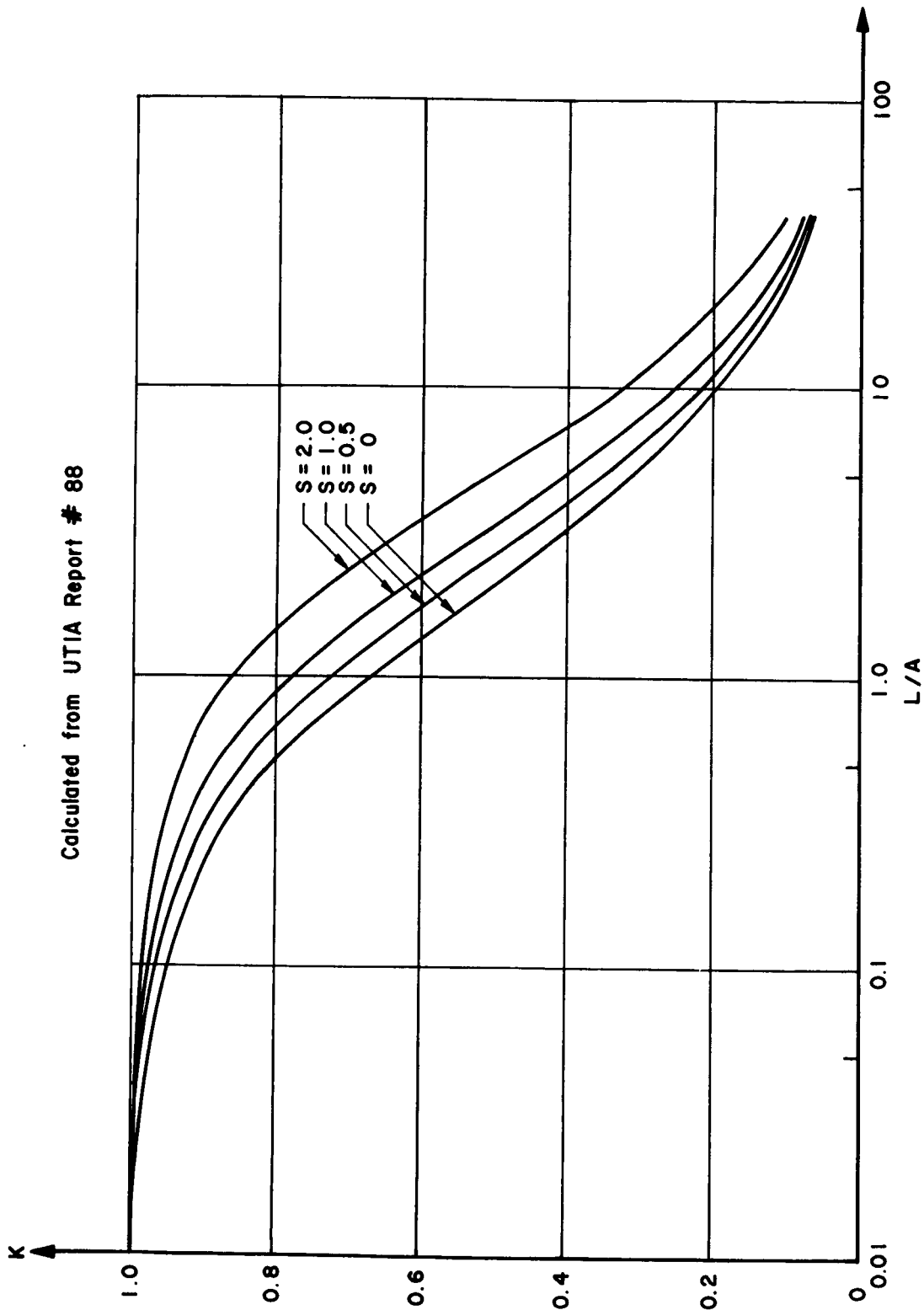


FIG. 8. TRANSMISSION PROBABILITIES FOR CYLINDRICAL DUCTS AS A FUNCTION OF LENGTH TO RADIUS RATIO,  $L/A$ , WITH SPEED RATIOS

Calculated from UTIA Report # 88

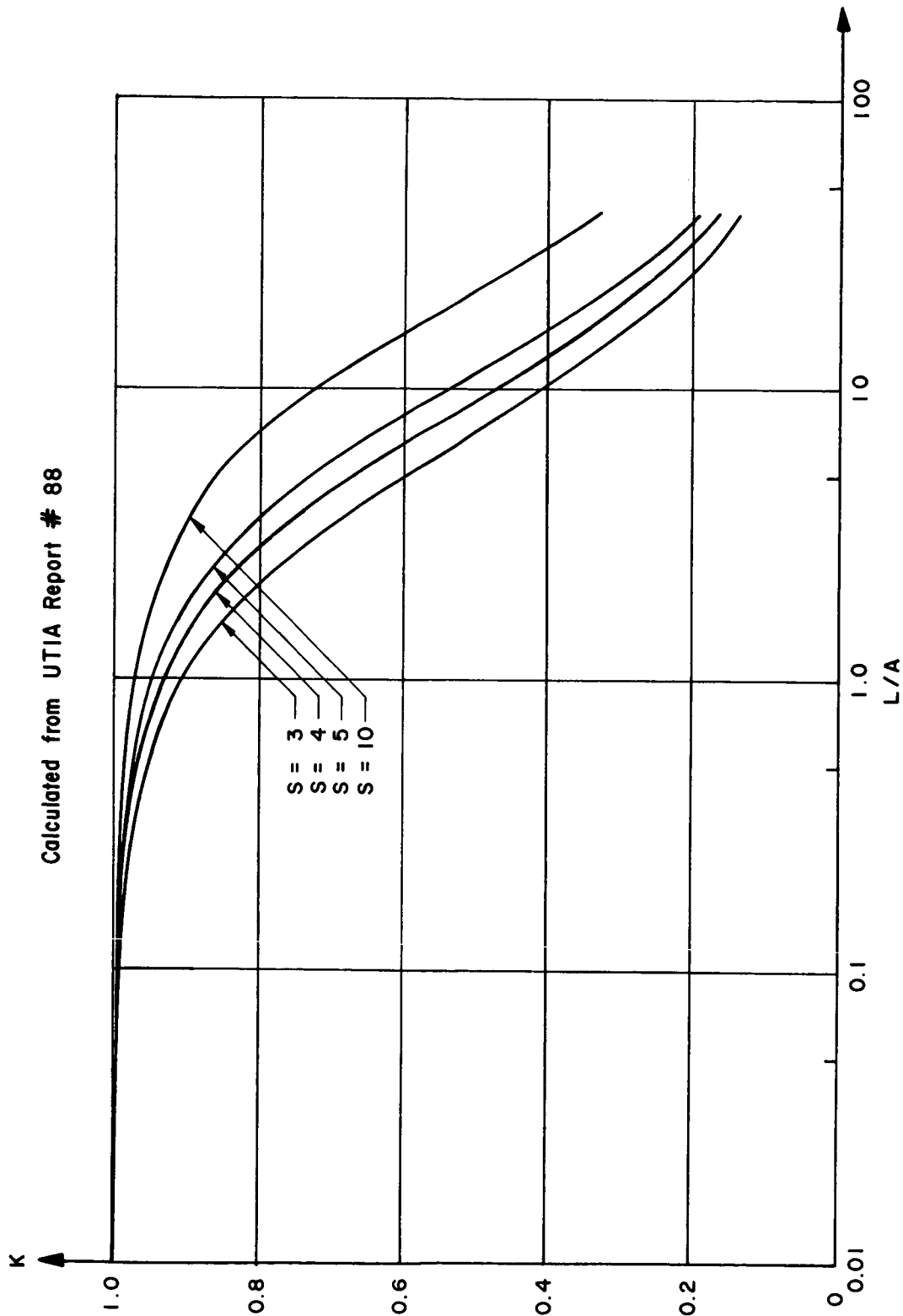


FIG. 9. TRANSMISSION PROBABILITIES FOR CYLINDRICAL DUCTS AS A FUNCTION OF LENGTH TO RADIUS RATIO,  $L/A$ , WITH SPEED RATIOS



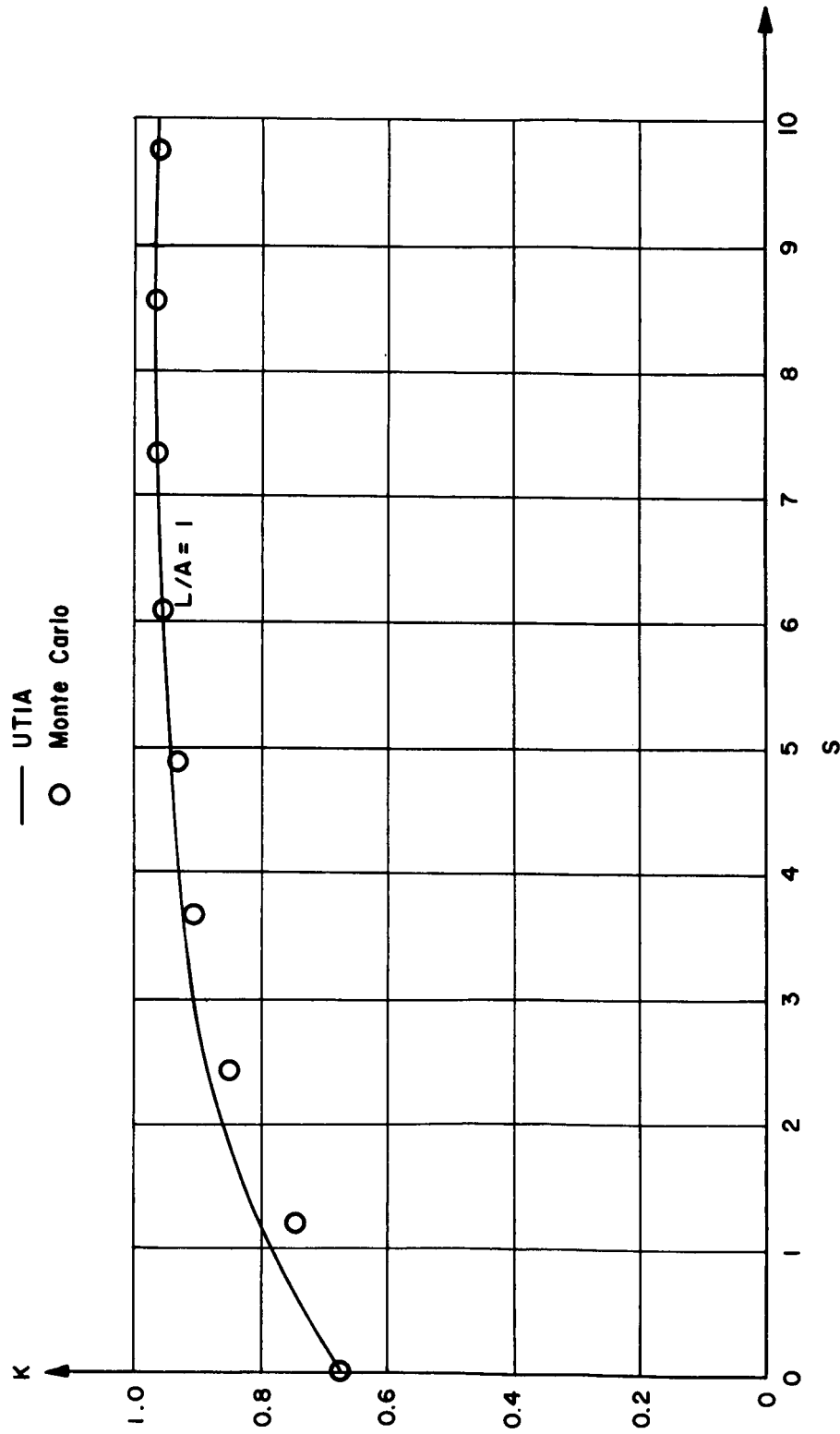


FIGURE 10. TRANSMISSION PROBABILITIES FOR CYLINDRICAL DUCTS AS A FUNCTION OF SPEED RATIO

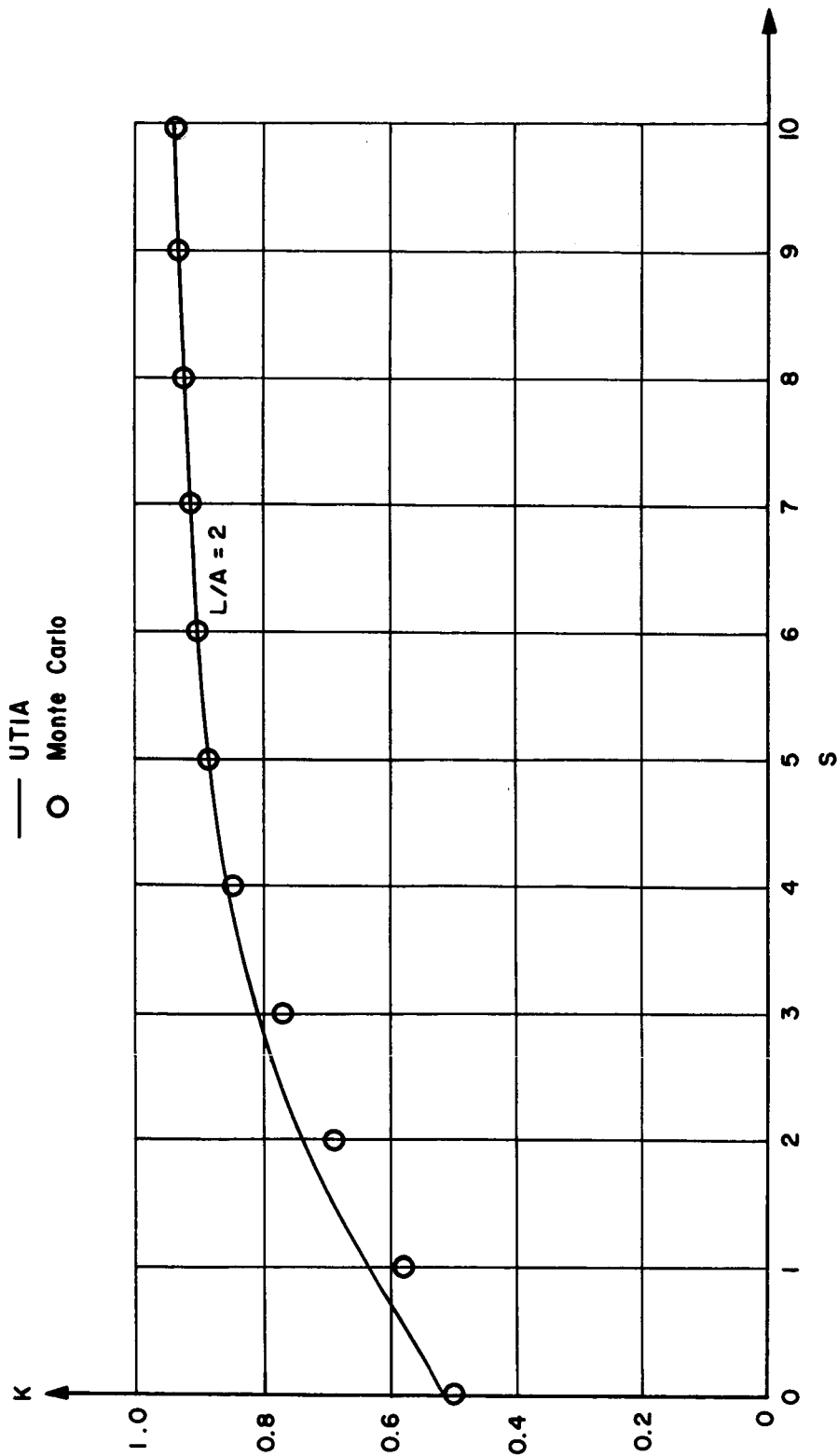


FIGURE II. TRANSMISSION PROBABILITIES FOR CYLINDRICAL DUCTS AS A FUNCTION OF SPEED RATIO

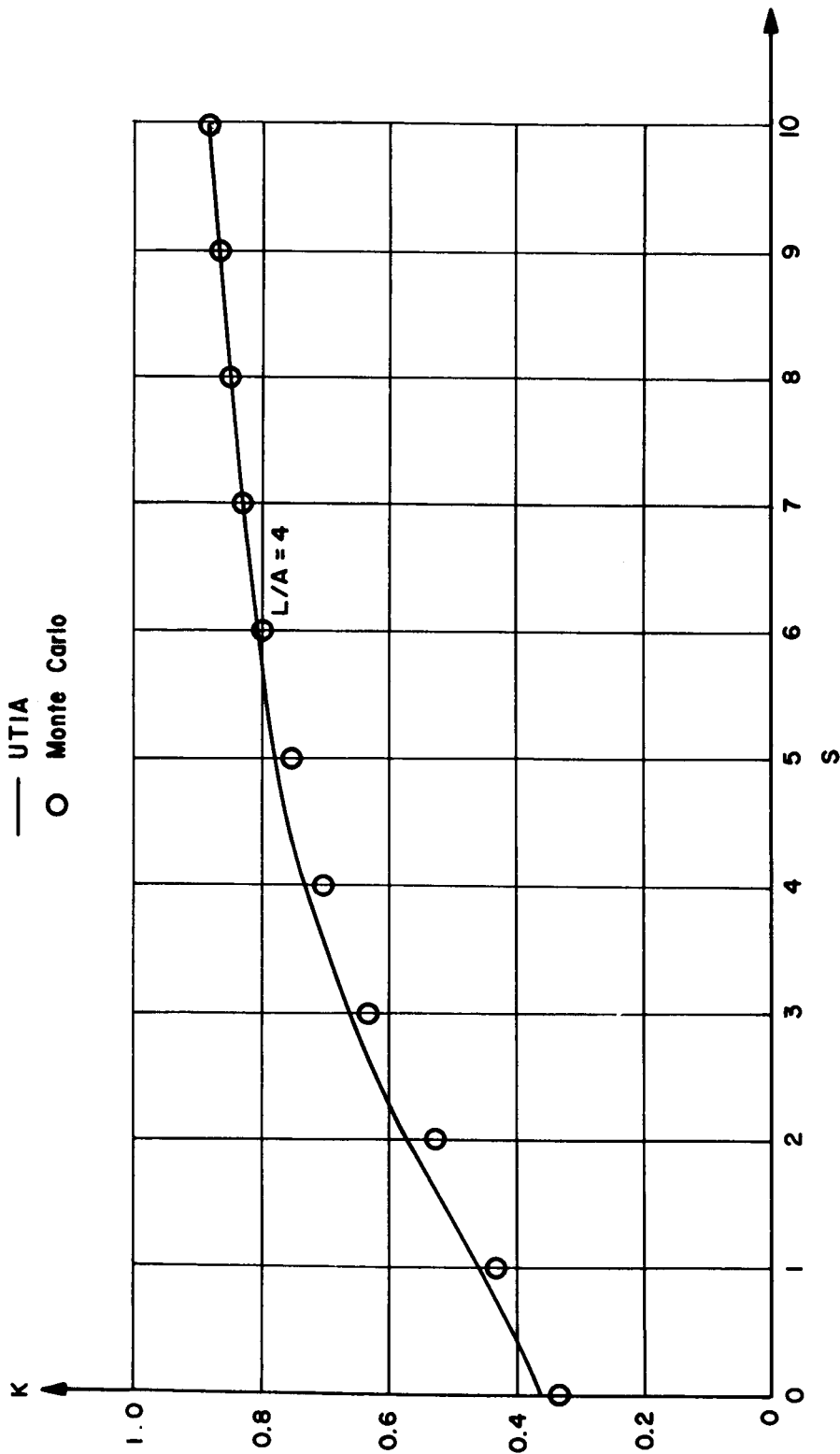


FIGURE 12. TRANSMISSION PROBABILITIES FOR CYLINDRICAL DUCTS AS A FUNCTION OF SPEED RATIO

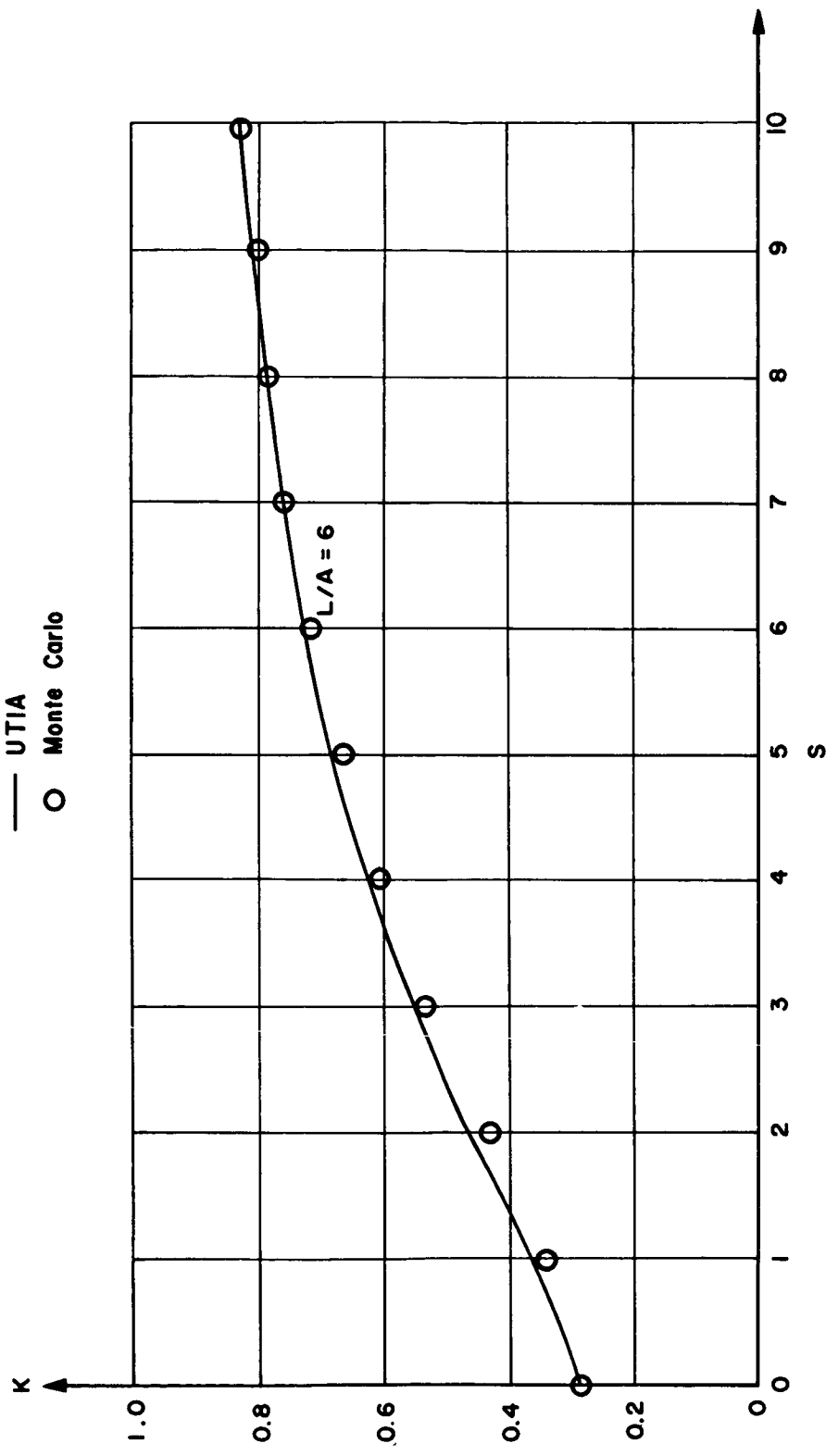


FIGURE 13. TRANSMISSION PROBABILITIES FOR CYLINDRICAL DUCTS AS A FUNCTION OF SPEED RATIO

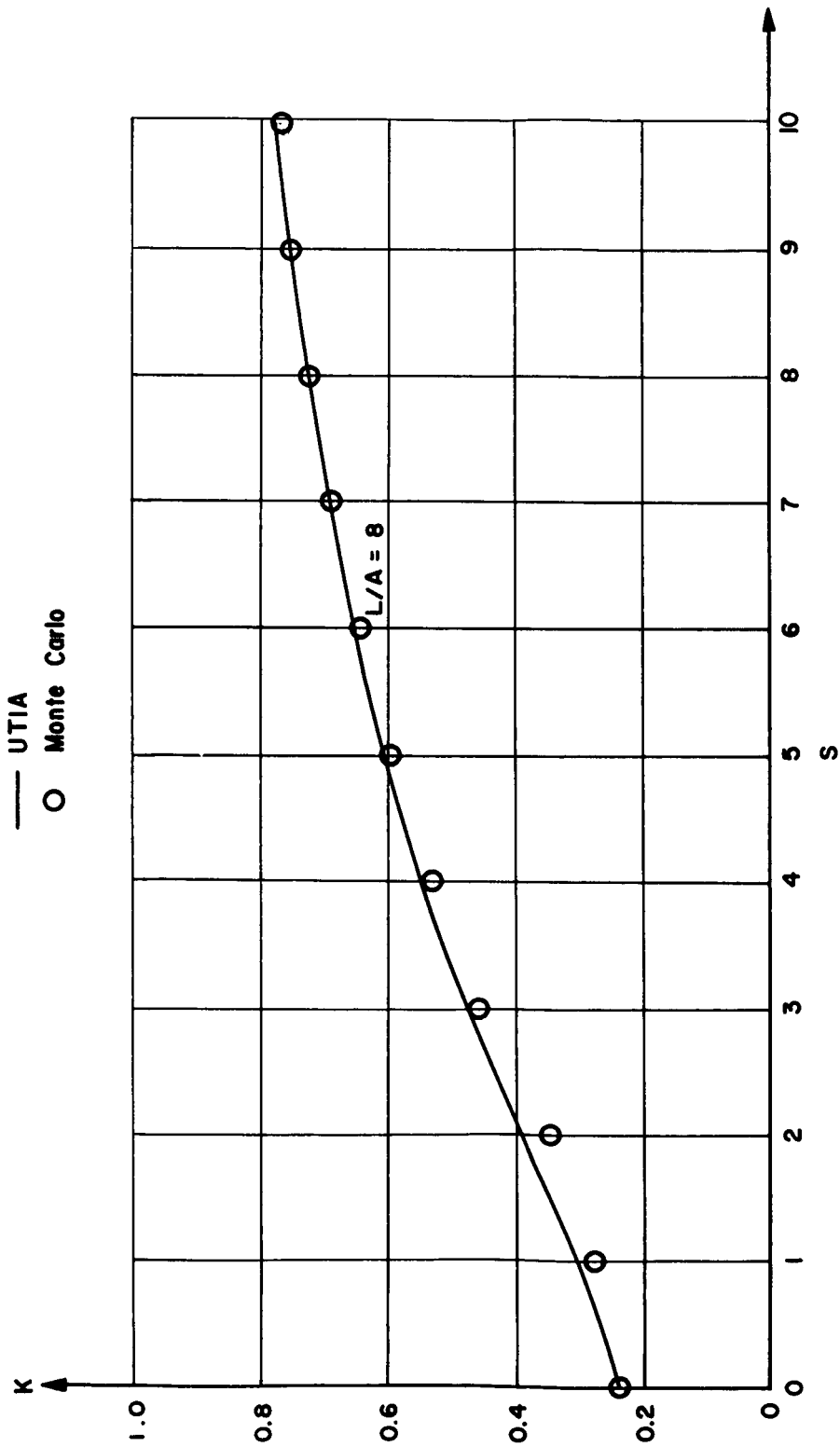


FIGURE 14. TRANSMISSION PROBABILITIES FOR CYLINDRICAL DUCTS AS A FUNCTION OF SPEED RATIO

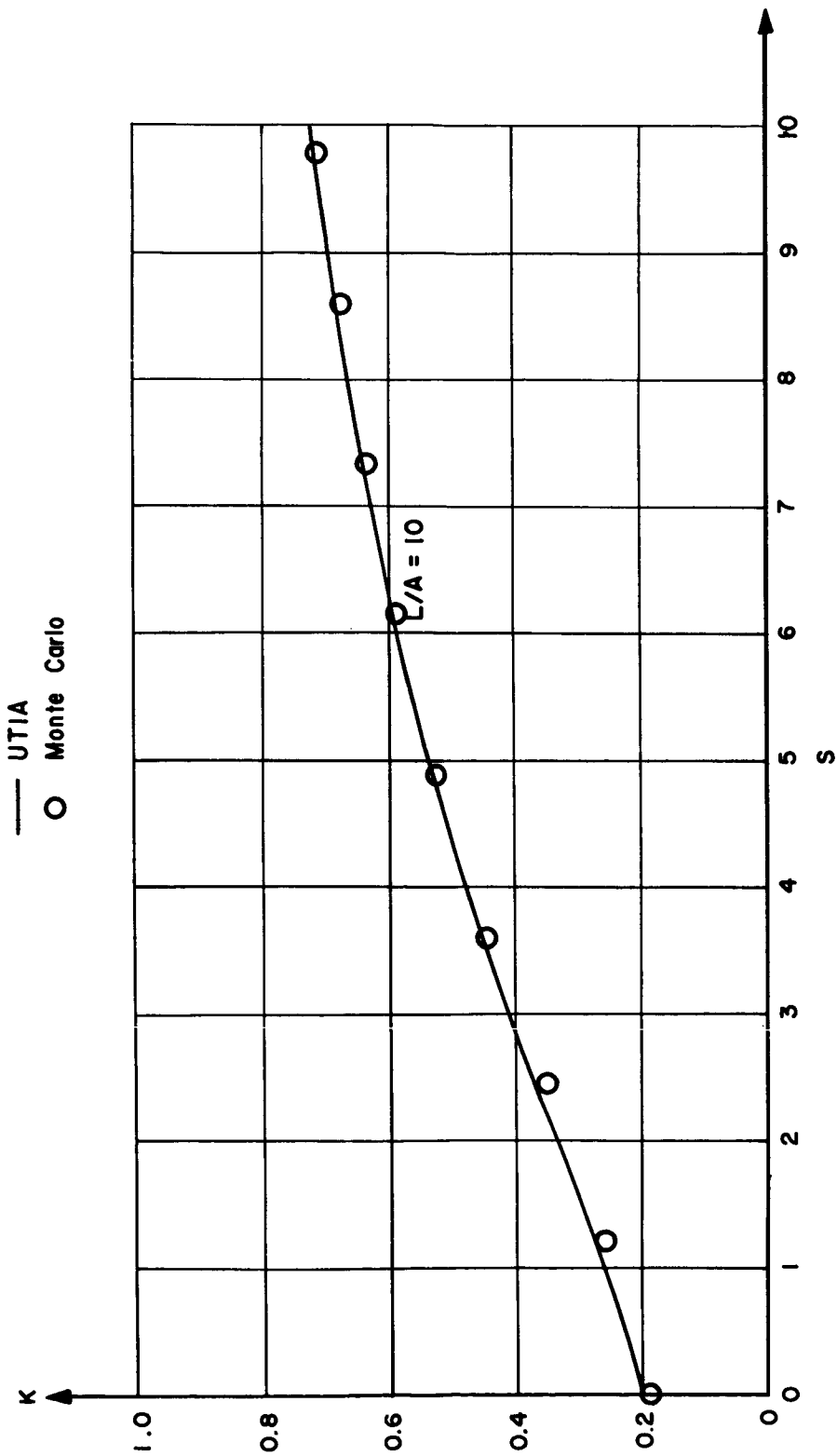


FIGURE 15. TRANSMISSION PROBABILITIES FOR CYLINDRICAL DUCTS AS A FUNCTION OF SPEED RATIO

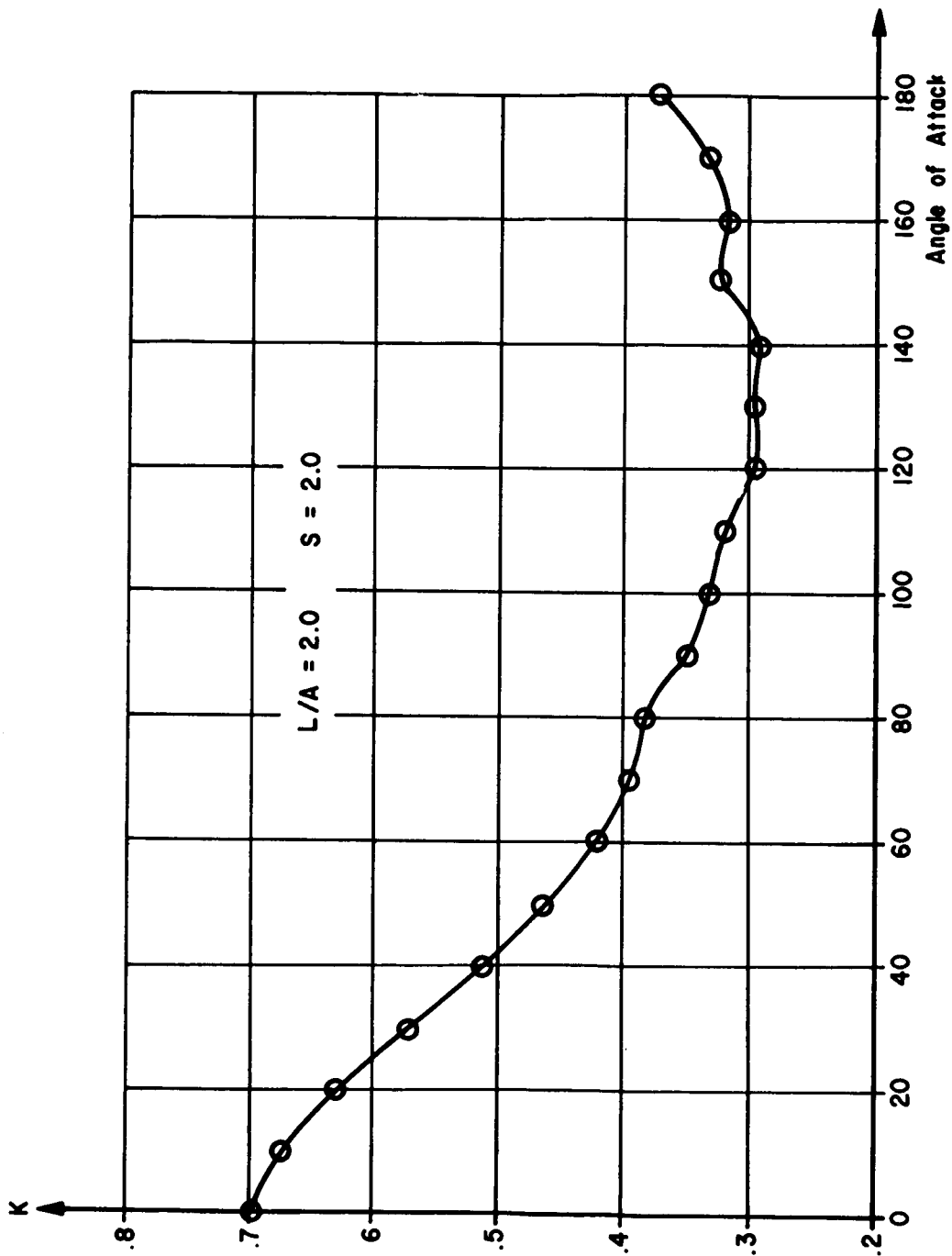


FIG. 16. TRANSMISSION PROBABILITIES FOR CYLINDRICAL DUCT AT A CONSTANT SPEED RATIO FOR VARIOUS ANGLES OF ATTACK

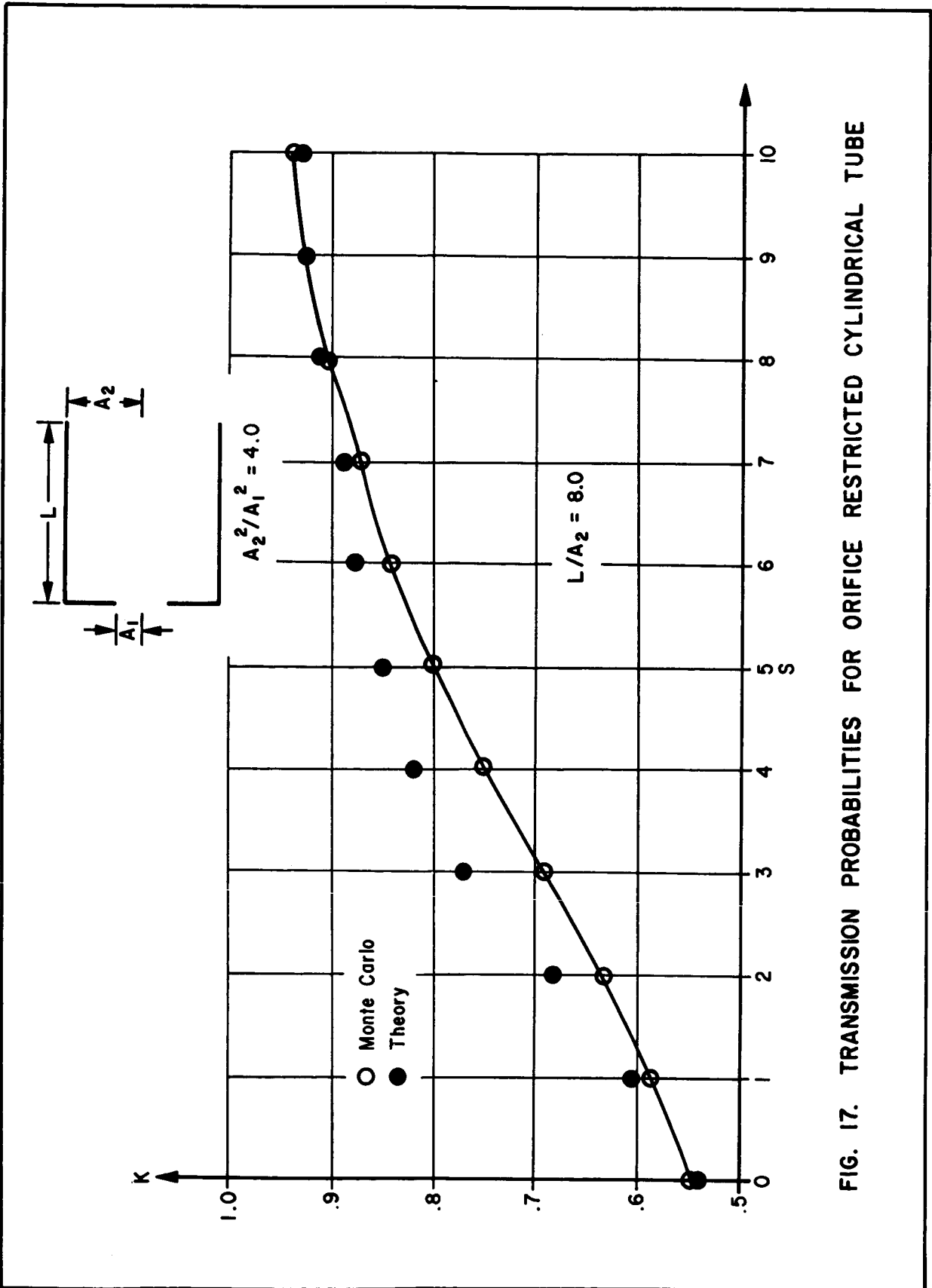


FIG. 17. TRANSMISSION PROBABILITIES FOR ORIFICE RESTRICTED CYLINDRICAL TUBE



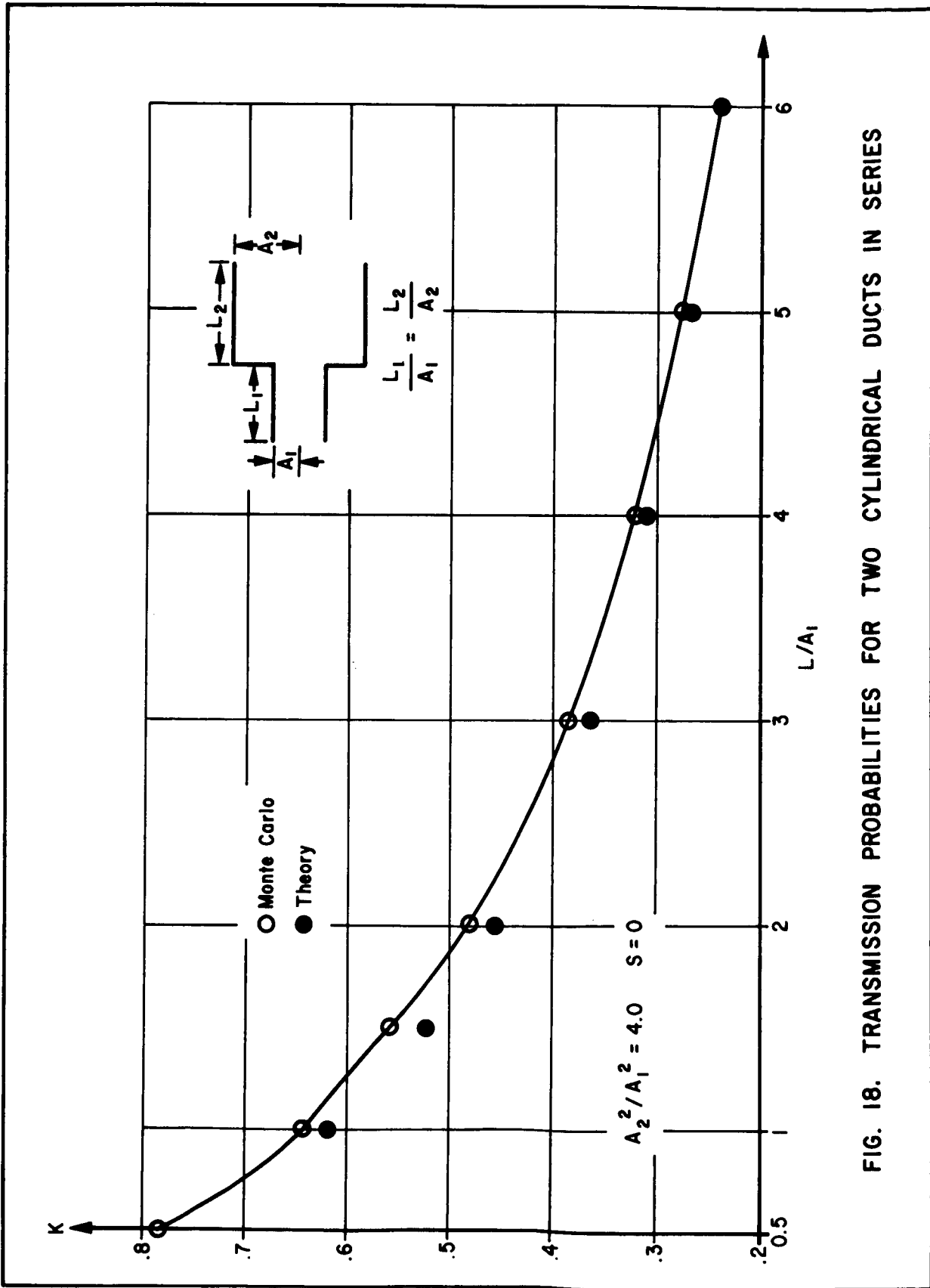


FIG. 18. TRANSMISSION PROBABILITIES FOR TWO CYLINDRICAL DUCTS IN SERIES

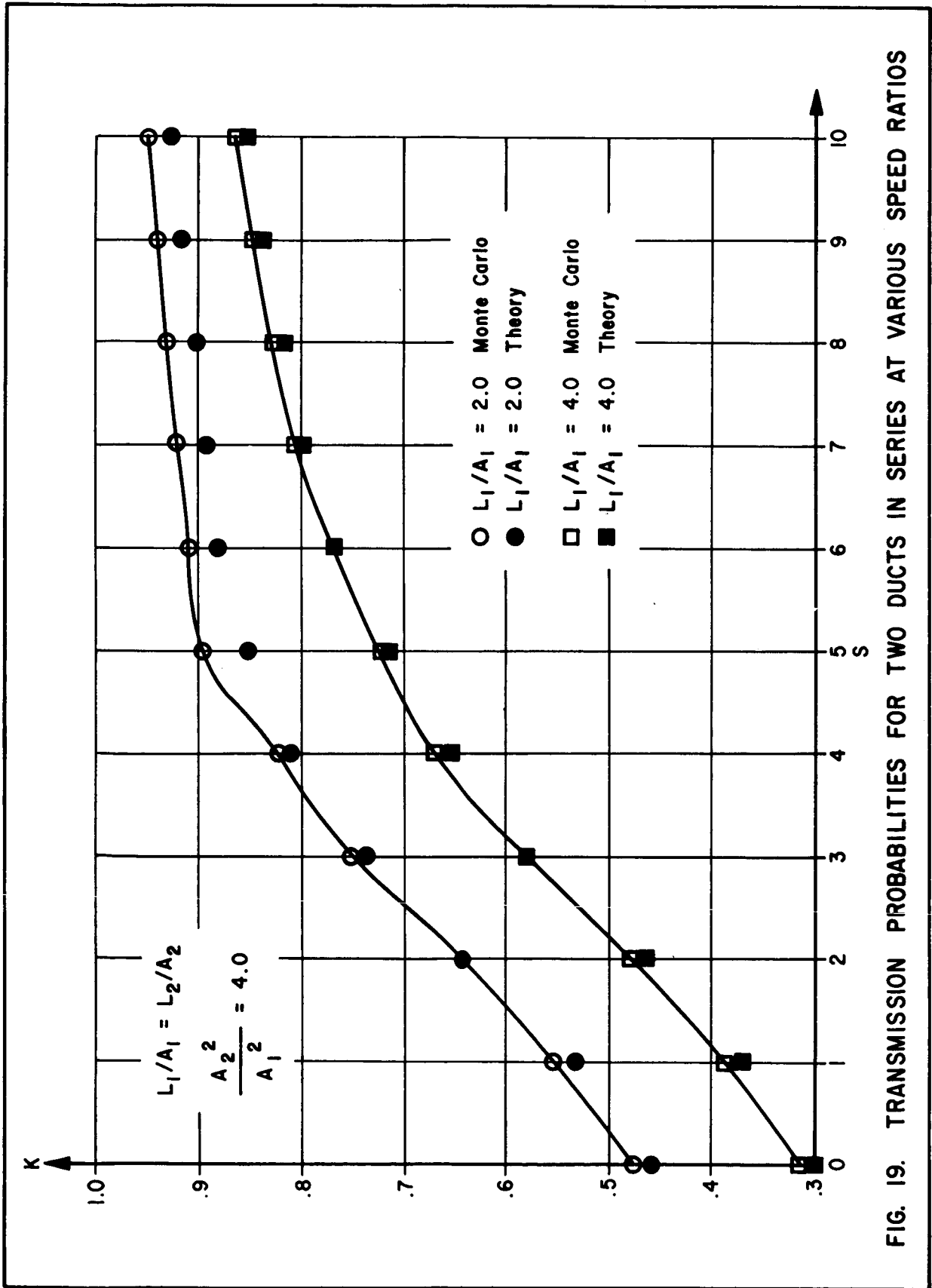


FIG. 19. TRANSMISSION PROBABILITIES FOR TWO DUCTS IN SERIES AT VARIOUS SPEED RATIOS

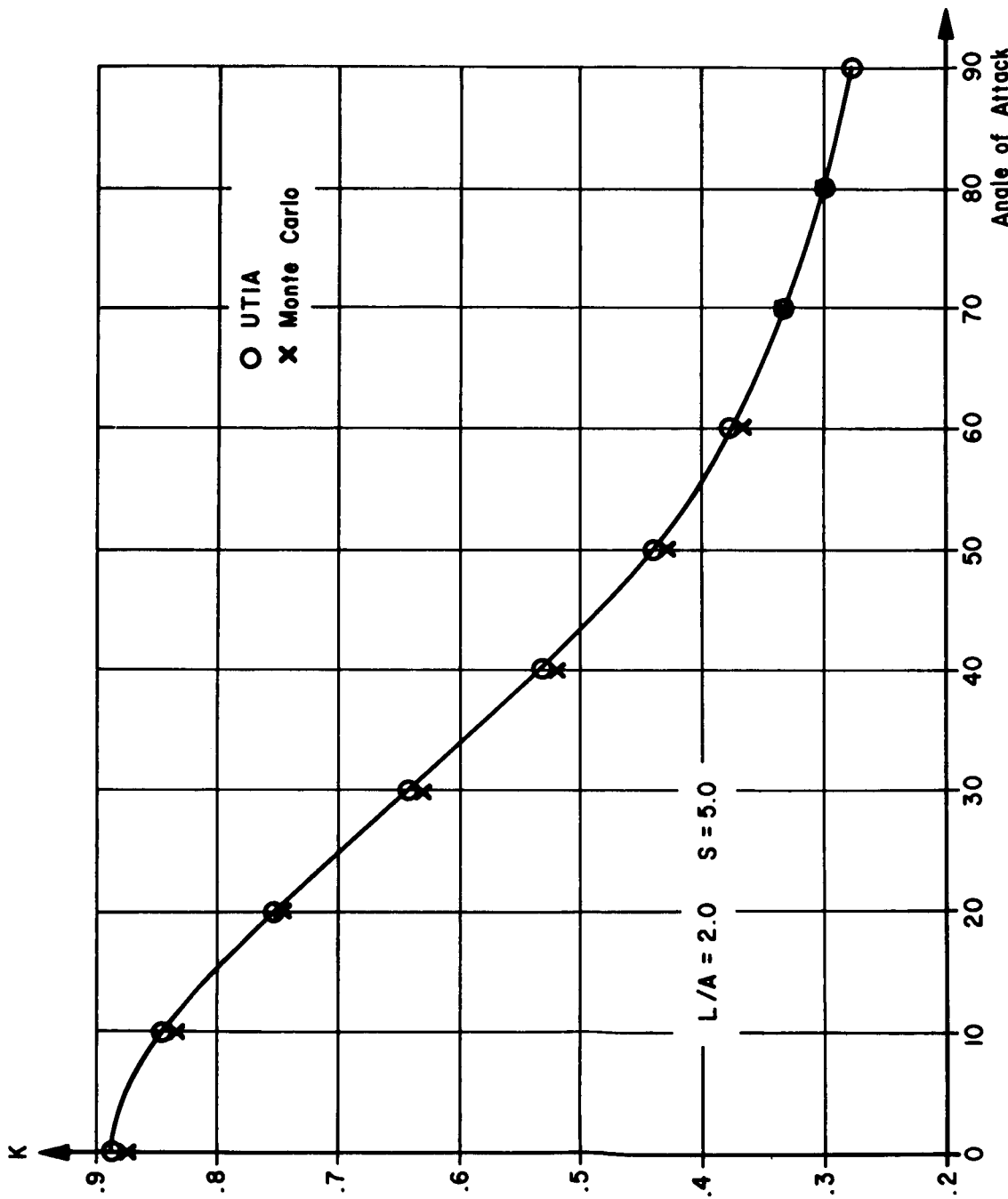


FIG. 20. TRANSMISSION PROBABILITIES FOR CYLINDRICAL DUCT AT A CONSTANT SPEED RATIO FOR VARIOUS ANGLES OF ATTACK

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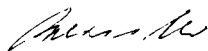
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