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HUNTSVILLE RESEARCH PARK
4800 BRADFORD DRIVE, HUNTSVILLE, ALABAMA

AUTOMATED NOSE FAIRING
DESIGN -- HONEYCOMB
SANDWICH CONSTRUCTION

Contract NAS8-15485

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9 November 1965

FOREWORD

This Technical Report describes one of three computer programs which were developed as tools for generating parametric weight and design data for nose fairings suitable for Saturn-class payloads. The work was performed by Lockheed Missiles & Space Company, Huntsville Research & Engineering Center, with support from the LMSC/Palo Alto Research Laboratories, for the National Aeronautics and Space Administration/Marshall Space Flight Center under Contract NAS8-15485, from July through November 1965.

The three computer programs developed under this contract are described in the following three reports.

1. Automated Nose Fairing Design -- Ring and Skin Construction, LMSC Technical Report LMSC/HREC A712552, November 1965.
2. Automated Nose Fairing Design -- Ring, Skin and Stringer Construction, LMSC Technical Report LMSC/HREC A712572, November 1965.
3. Automated Nose Fairing Design -- Honeycomb Sandwich Construction, LMSC Technical Report LMSC/HREC A712573, November 1965.

Many of the subroutines and the methods of specifying external geometry and aerodynamics loads are common to all three programs.

This report describes the computer program for honeycomb sandwich construction. Major contributors to the development of this computer program are B. O. Almroth of the Palo Alto Research Laboratories and E. S. Hendrix, I. M. Landis, and Z. Adams of Huntsville Research & Engineering Center. Appendix K of this report was written by B. O. Almroth, and the remainder was written by Z. Adams.

SUMMARY

The computer program described in this report synthesizes near-optimum designs for honeycomb sandwich nose fairings with an external geometry consisting of up to ten intersecting conical frustums (or cylinders) capped with a spherical nose cap. The combined effects of bending moments, axial loading due to drag, and lateral pressure are considered in performing the design. Either standard gauges or non-standard gauges can be used for faces and ribbons.

The main part of the report is devoted to description of the logic followed in designing a fairing, description of the computer program, and instructions on operation of the program. The program listing and details of the methods of analysis used in design of fairing appear in the Appendixes.

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INTRODUCTION

The computer program described in this report synthesizes near-optimum structural designs for ring stiffened, honeycomb sandwich nose fairings. When the external geometry, aerodynamic loading, ring spacing, and a practical set of design constraints are given, the computer program selects a combination of honeycomb panel face thickness, core height and ring cross-section that gives a minimum fairing weight. Provisions have been made to use either standard or non-standard gauges of material for faces and ribbons. The external geometry can consist of up to ten intersecting conical frustums capped with a spherical nose cap. Figure 1 illustrates the external geometry and type of construction.

A condensed flow chart illustrating the major logical steps performed by the program is shown in Figure 2. Design begins at the base of the fairing and moves toward the nose cap. Each bay is designed to withstand loads imposed by the interaction of bending moments, axial loading, lateral pressure and/or internal pressure. The combination of face thickness, core height, ribbon thickness, cell size and ring cross-section which results in minimum weight-to-volume ratio for a bay is considered to be the optimum design for that bay.

The computer program consists of the main program and a number of subroutines. The logical steps needed to design the fairing are performed in the main program. Specialized and repetitive functions are performed in the subroutines. This particular arrangement, as well as the liberal use of comment cards, is intended to facilitate future modifications.

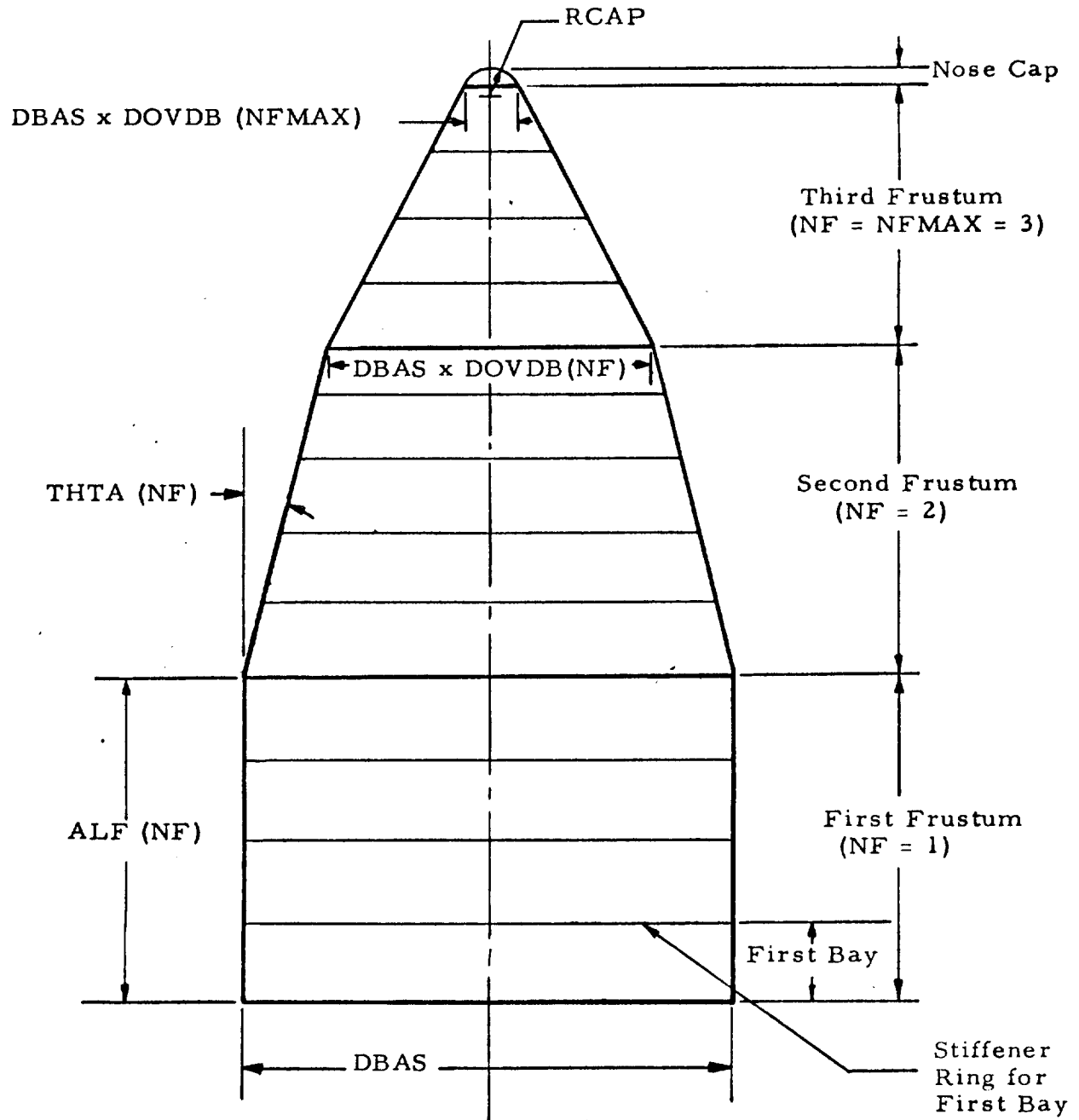
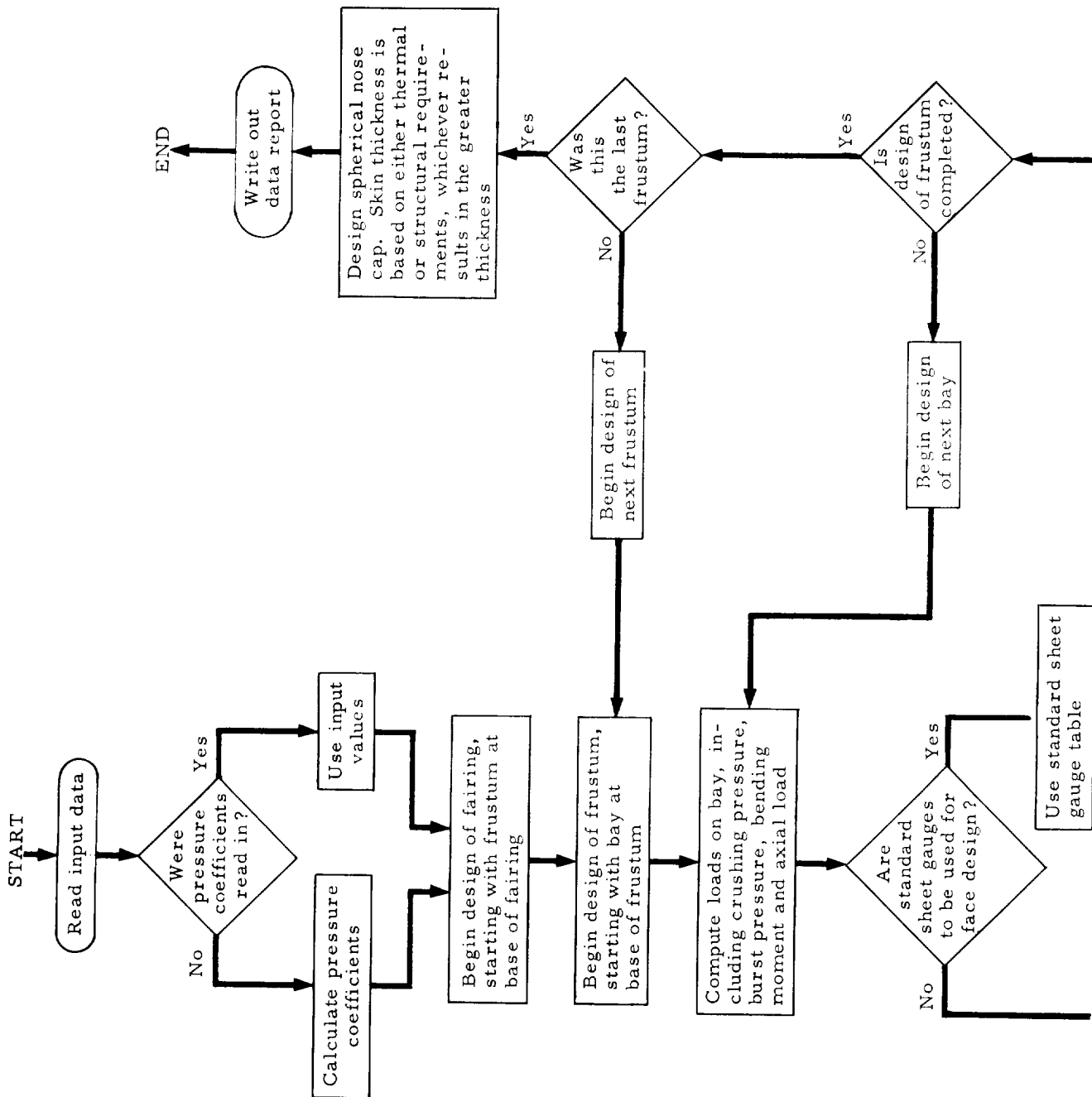


Figure 1 - Nose Fairing Geometry



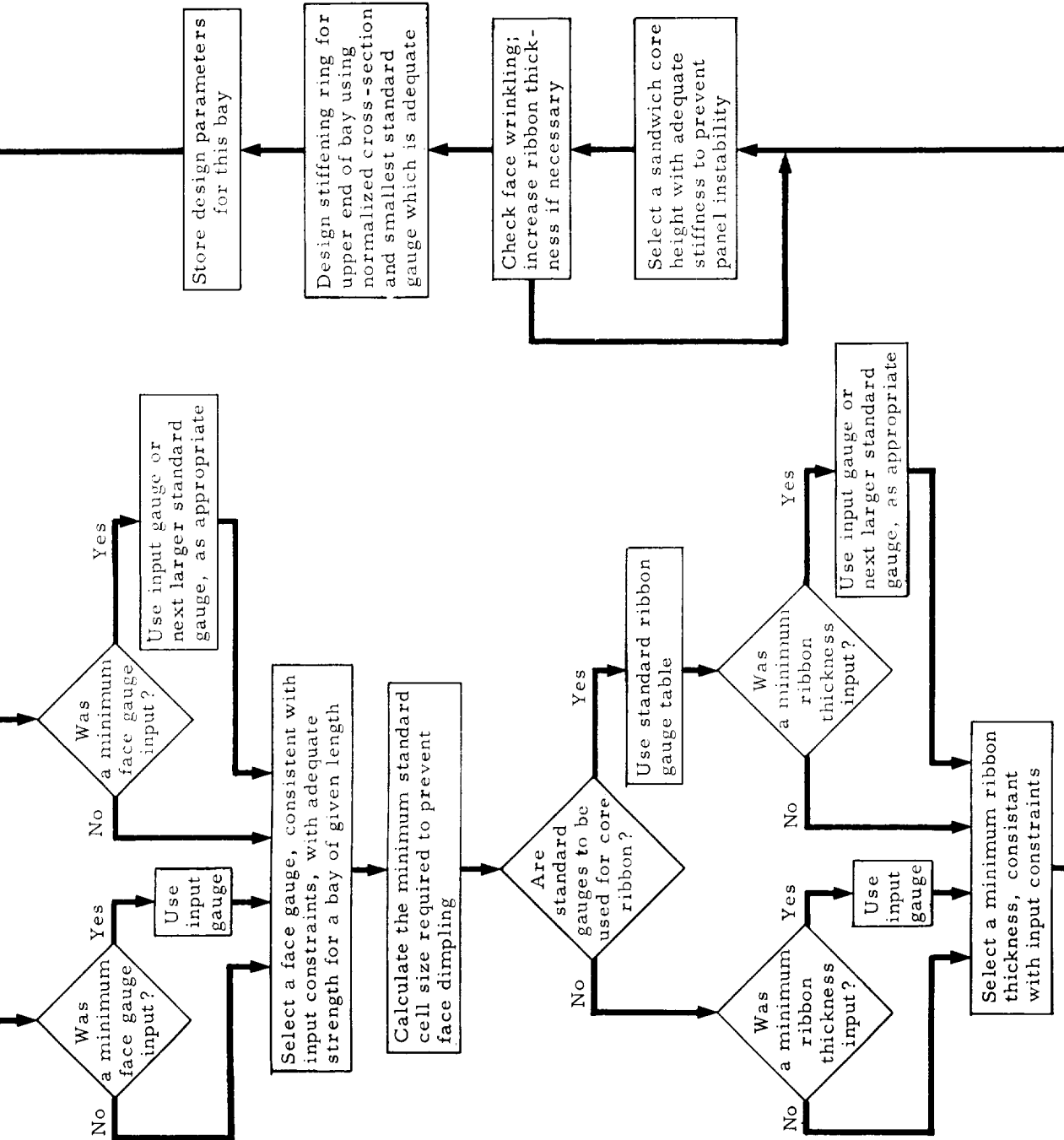


Figure 2 - Flow Chart Showing the Major Steps Performed in Optimizing the Design of a Multi-Frustum Nose Fairing - Honeycomb Sandwich Construction

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C      AXIS. IF THE CELL HEIGHT TO WIDTH RATIO IS LESS THAN 5.0. SWD0347
C      APPLY A WARPAGE CORRECTION FACTOR BASED ON FIGURE 6 OF THE SWD0348
C      PENZIEN AND DIDRIKKON PAPER. SWD0349
      PHI = 0.5 * PI SWD0350
      GC2 = (GORE * TRIBN * SBETA * (R + CBETA)) / (A * ((1. + R) SWD0351
      1 * SBETA**2 * (COS(PHI))**2) + (R + CBETA)**2 * (SIN(PHI))**2)) SWD0352
      PHI = 0.0 SWD0353
      GC1 = (GORE * TRIBN * SBETA * (R + CBETA)) / (A * ((1. + R) SWD0354
      1 * SBETA**2 * (COS(PHI))**2) + (R + CBETA)**2 * (SIN(PHI))**2)) SWD0355
      KY = 0 SWD0356
      MAKE FIRST ESTIMATE OF CORE HEIGHT. SWD0357
      H = .25 + TF SWD0358
      IF (1.GT.1) KY = 1 SWD0359
      T1 = TF SWD0360
      T2 = TF SWD0361
251 CONTINUE SWD0362
      IF ( H/W .GT. 5.0) GO TO 228 SWD0363
      Y = (9.3 / (H/W)) - (2.1 / (H/W)**2) SWD0364
      IF ( H/W .LT. 0.5) Y = 11.0 SWD0365
228 CONTINUE SWD0366
      G1 = GC1 * (1.0 + .01 * Y) SWD0367
      G2 = GC2 * (1.0 + .01 * Y) SWD0368
      P = PDESMX SWD0369
      CHECK PANEL STABILITY ON WINDWARD SIDE. IF DESIGN IS SWD0370
      SATISFACTORY, PROCEED TO CHECK LEEWARD SIDE. IF DESIGN IS SWD0371
      UNSATISFACTORY, INCREASE CORE HEIGHT. IF A HEIGHT OF 4 INCHES SWD0372
      IS REACHED AND THE DESIGN IS STILL UNSATISFACTORY, INCREASE SWD0373
      FACE THICKNESS. SWD0374
      CALL INSTBL SWD0375
      IF (KG .EQ. 1) GO TO 98 SWD0376
      IF (CRG .GE. ANFINN) GO TO 252 SWD0377
      IF ( H .GT. 2.0) GO TO 229 SWD0378
      H = H + .125 SWD0379
      KY = 1 SWD0380
      GO TO 251 SWD0381
229 CONTINUE SWD0382
      IF (KTFSTD .EQ. 1) GO TO 234 SWD0383
      TF = 1.1 * TF SWD0384
      GO TO 239 SWD0385

```

TECHNICAL DISCUSSION

1.0 THE MAIN PROGRAM

1.1 Terminology and Geometric Parameters

Following are definitions of terms used in this technical discussion. The terms are also illustrated in Figure 1.

- base - the larger end of the bay, frustum or fairing
- bay - the conical frustum between two rings plus the ring at the upper end of this conical frustum
- frustum - the conical frustum of fairing consisting of all bays having the same half angle
- nose cap - the spherical segment which closes the top of the fairing.

The external geometry of the fairing is specified by the base diameter of the fairing (DBAS), by the half-angle of each frustum (THETA (NF)), and either the ratio of the top diameter of each frustum to the base diameter of the fairing (DOVDB (NF)), or (mandatory when the frustum is a cylinder) the length of the frustum (ALF (NF)). Frustums are numbered by index NF starting at the base of the fairing. Frustum geometry is completely described by the same parameters used to describe fairing geometry. Bay geometry is described by the base diameter of the bay (DSUBB), the half angle of the bay (THETA), and the length of the bay measured along the axis of symmetry (ALB). Bays within a frustum are numbered by index I starting at the base of the frustum. The outside diameter of a ring associated with a bay is equal to the top diameter of the bay. All dimensions of the ring cross-section are expressed in terms of the material thickness used to form the ring (see Figure 3). This thickness will be one of the standard gauges stored in the program.

1.2 Design Logic

The major logical steps followed in designing a nose fairing are shown in the condensed flow chart in Figure 2. In Appendix A, more detailed information is provided by a program listing which includes detailed comments describing in words the operations being performed. Definitions of the more commonly used variable names in this listing appear in Appendix B. The discussion which follows is supplemental to the information in the flow chart and program listing, and in general follows the sequence of the listing.

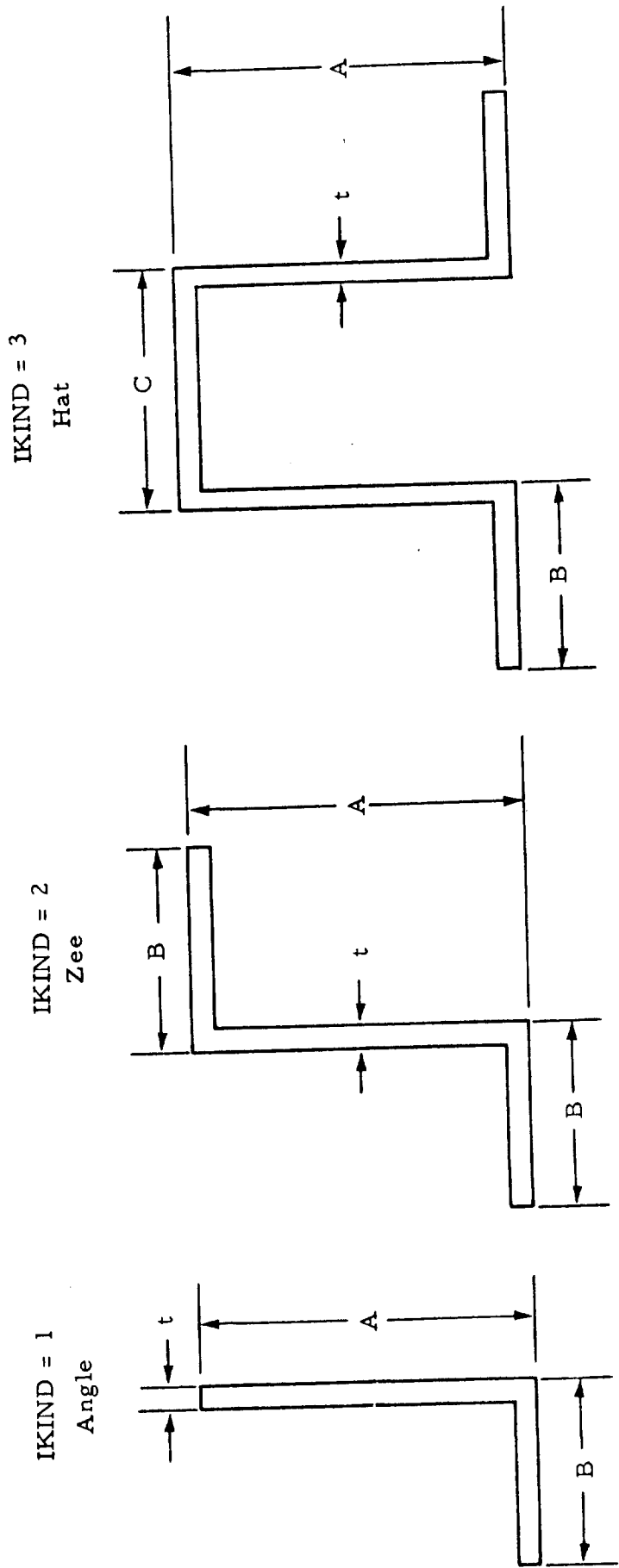


Figure 3 - Stiffener Cross-Section

1.3 Required Input Data

The following input data is required by the computer program:

1. External fairing geometry (Figure 1)
2. Design constraints
 - a. Minimum distance between rings, in.
 - b. Ring type (zee, hat, or angle), also flange and web thickness ratios - see Figure 3.
 - c. Ring outstanding flange buckling stress level, psi.
3. Structural material
 - a. Kind of material to be used. (Certain properties of five materials are stored in the program. See Section 1.5 of Technical Discussion.)
 - b. Additional properties of sandwich face material
 - (1) Tensile allowable stress, psi
 - (2) Compressive allowable stress, psi
 - (3) Shear allowable stress, psi
 - (4) Young's modulus, psi
4. Aerodynamic data at a design point in the trajectory
 - a. Mach number
 - b. Dynamic pressure
 - c. Angle of attack
 - d. Difference between internal pressure of fairing and free-stream pressure
5. Factor of safety (If no value is input, a factor of 1.4 will be used.)
6. Program controls
 - a. Is pressure profile data input? If so, the type of lift data is indicated.
 - b. The desired type of output is indicated. (See Section 1.13.)
7. Pressure profile data (optional). If a pressure profile is not input, it is computed in Subroutine AERO.

Detailed instructions on how to input this data are provided in Section 3.

1.4 The Pressure Profile

Whether input or computed, the system used to specify the pressure profile in the axial direction on the section of fairing composed of conical frustums is illustrated in Figure 4. LT is an index indicating station number, starting with the first station at the junction of the nose cap and top frustum. Uniform spacing between stations is not necessary. Two stations must be located at each intersection of the conical frustums. Where discontinuities in the pressure profile exist, two stations can be indicated for the same location.

The following three parameters are required by the computer program at each station.

1. CPO (LT) - The pressure coefficient at zero angle of attack
2. CPA (LT) - The difference between the pressure coefficient on the windward side when flying at an angle of attack and CPO (LT)
3. XOD (LT) - The axial distance measured from the tip of the nose cap divided by the fairing base diameter

When the pressure profile is input to the program, three options are available for inputting lift data.

1. CPA (LT) as described above
2. CPA (LT) per radian angle of attack
3. $\frac{\partial}{\partial (x/D)} \left(\frac{\partial C_N}{\partial \alpha} \right)$

In Option 3

C_N = the normal force coefficient with the fairing base as a reference area

α = angle of attack in radians

x/D = distance from the leading point in calibers

After they are read into the program, the lift data in Options 2 and 3 are converted to the form of Option 1. A sinusoidal pressure distribution (see Figure 5) in the circumferential direction is used in converting Option 3 to Option 1. Provisions can be made to read in other types of lift parameters, if the parameter can be converted to CPA (LT) after it is input.

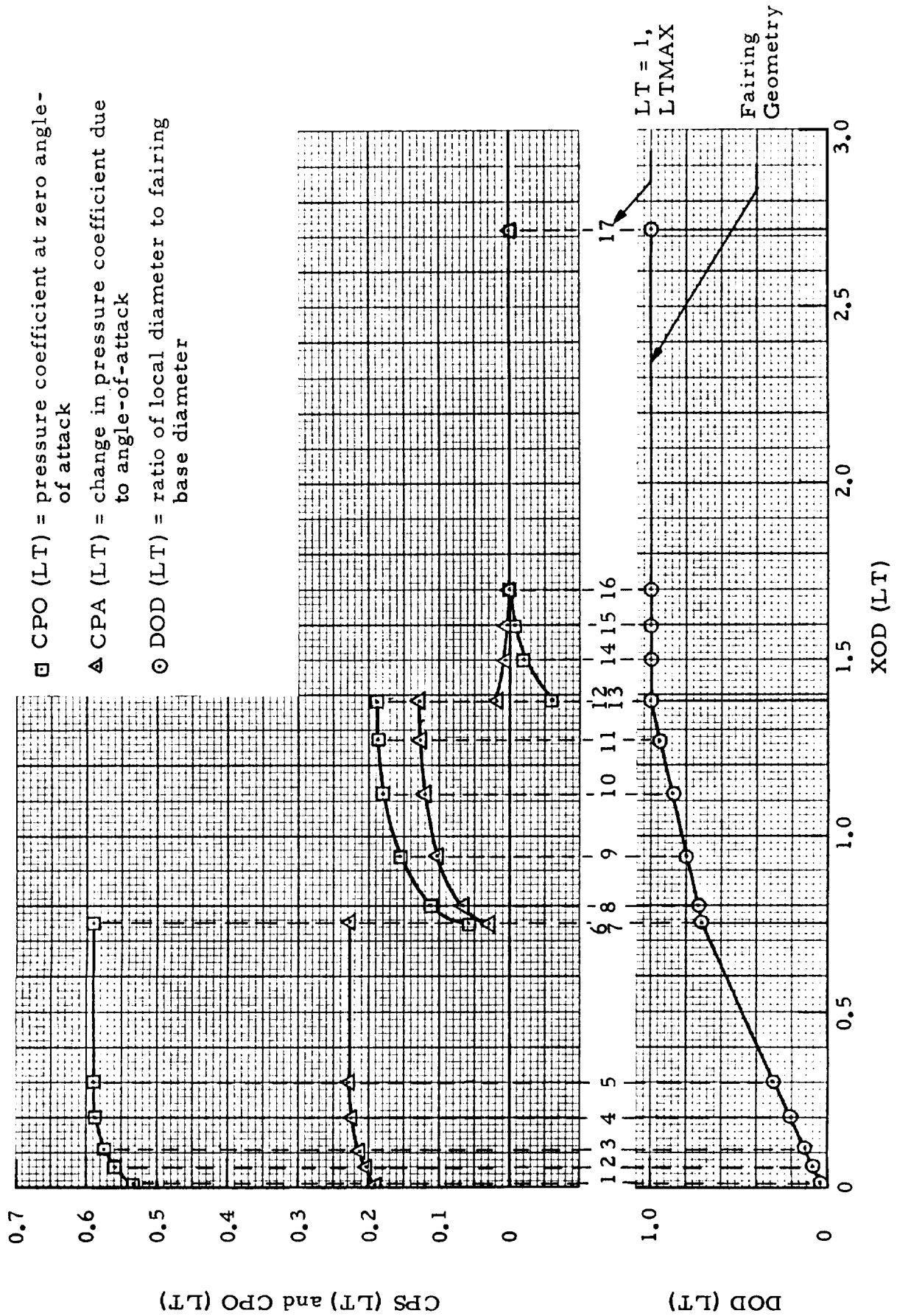


Figure 4 - Pressure Profile Used in Sample Problem

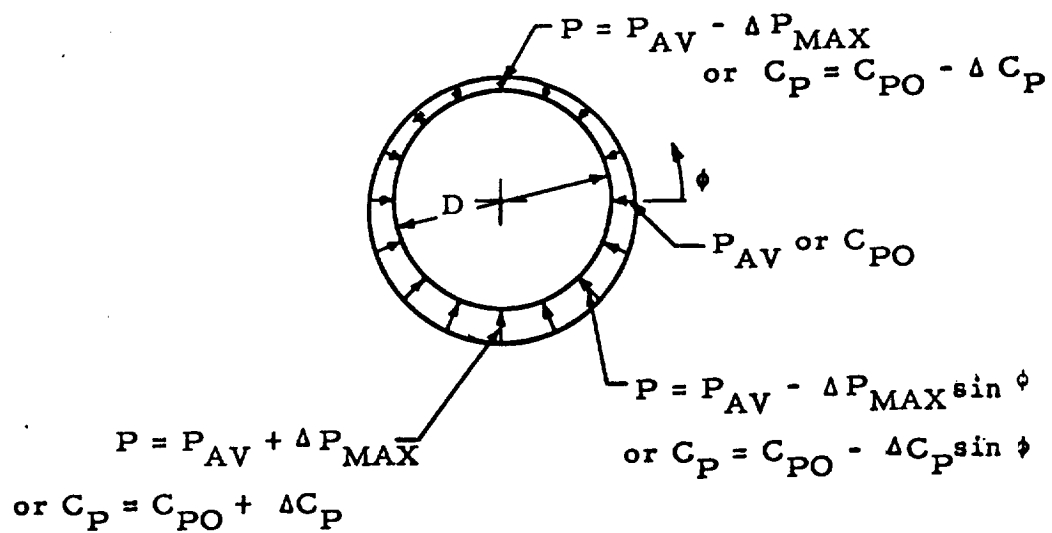


Figure 5 - Circumferential Pressure Distribution

The option is available to either compute or input axial force and lift data for the nose cap. In either case they are specified by the following three parameters:

1. Drag coefficient with the base area of the nose cap as the reference area
2. Normal force coefficient per radian angle-of-attack with the base area of the nose cap as a reference area
3. The location of the center of lifting pressure measured from the base of the nose cap

1.5 Material Properties

Properties for the following five materials are now stored in the program in Subroutine PROPTY:

1. Aluminum
2. Magnesium
3. Titanium
4. Stainless steel
5. Lockalloy

Additional materials can be readily added to this list. Properties which are stored are as follows:

1. Modulus of elasticity
2. Poisson's ratio
3. Density
4. Maximum allowable temperature

If a value is input for maximum allowable temperature the stored value is not used.

The quantities stored in the program are sufficient for most of the program's operations. However, additional data is required for the shell buckling analysis. Since the shell buckling analysis is based upon elastic behavior, it is necessary to input a face cut-off stress level which will ensure that plasticity effects do not become significant.

1.6 Thermal Considerations

The maximum temperature reached by the skin due to aerodynamic heating can be controlled by placing a lower limit on the skin thickness to be used in the nose cap and the top frustum. These minimum thicknesses are computed in Subroutine THERML. The minimum thickness computed for the nose cap is based on heating at the stagnation point, and the minimum

thickness for the top frustum is based on heating at the point at which the nose cap is tangent to the top frustum. The equations in Subroutine THERML are based on a nominal trajectory of two-stage Saturn V vehicle ascending to a 100 nautical mile circular orbit.

1.7 Standard Skin Gauges

Only standard gauges are used in designing stiffening rings. Either standard or non-standard gauges may be used for the sandwich faces and ribbons.

1.8 Design Loads

The individual bay is subjected to bending moments, axial forces and lateral pressure loads. Bending moments and axial forces at the base of the bay are computed in Subroutine LOAD, using the pressure profile data. These loads are then converted to line loads (force per unit of length on the circumference) on both the windward and leeward sides of the bay, and the factor of safety is applied.

The lateral pressure used in design of the bay is the difference between external and internal pressure multiplied by the factor of safety.

1.9 Last Bay in the Frustum

Before beginning the design of a bay, a check is made to determine if there is sufficient length remaining on the frustum for one more bay of minimum length. If there is not sufficient length for one more bay, the length of the last bay designed is added to the remaining undesigned length.

Additional weight is added to the top ring of each frustum to provide for attachment to the next frustum or nose cap.

1.10 Nose Cap Design

Both structural and thermal requirements are considered in the design of the nose cap. The thickness required to limit the temperature to the specified maximum is computed in Subroutine THERML. (For details of the thermal analysis see Appendix J.) The thickness required to withstand aerodynamic loads is computed in Subroutine TNOSST. (For details see Appendix H.) In both cases thickness is determined for conditions at the stagnation point. The greater of these two thicknesses is then used to design an unstiffened cap with uniform skin thickness. Nose cap skin thickness is not limited to standard gauges.

1.11 Output Data

Three options are available on the amount of detail provided by the output data.

1. Design summary only (Figure 6A)

2. Design summary plus design details (Figures 6A, 6B and 6C)
3. Design summary plus design details plus loads details (Figures 6A, 6B, 6C, 6D and 6E).

Most of the headings appearing in the output are self-explanatory. However, there are two which require some comment.

- Weight Index - Weight of the bay divided by the inclosed volume of the bay.
- Line Load - Force per running inch of circumferences, parallel to the skin surface, normal to the circumferential direction. A factor of safety has been applied to this force.

DATA REPORT FOR OPTIMIZED NOSE FAIRING STRUCTURE
CASE NUMBER 1

MATERIAL = ALUMINUM

AERODYNAMIC LOADS

DYNAMIC PRESSURE, LBS./SQ. FT. QBAR = 765.00
MACH NUMBER AT DESIGN DYNAMIC PRESSURE AMACH = 1.500
ANGLE OF ATTACK AT DESIGN DYNAMIC PRESSURE, DEGREES ALPHA = 4.50
INTERNAL-TU-AMBIENT PRESSURE DIFFERENCE AT DESIGN CONDITIONS, PSI DELTAP = -0.000
AMBIENT PRESSURE AT DESIGN CONDITIONS, PSI PSTAT = 3.373

CONSTRAINTS ON DESIGN OF FRUSTUMS

MAXIMUM ALLOWABLE TEMPERATURE OF SKIN, DEG. F. TMPMAX = 10000.0
CRITICAL SECTION SKIN THICKNESS REQUIRED FOR THERMAL REASONS, IN. TCBNTH = 0.0000

DESIGN SUMMARY FOR FRUSTUM SECTION

FRUS- TUM NO.	LARGE DIA. (IN.)	SMALL DIA. (IN.)	HALF ANGLE (DEG)	LENGTH (IN.)	MIN. RAY LENGTH (IN.)	MIN. GAUGE (IN.)	NO. OF BAYS	USEFUL VOLUME (CU. FT.)	WEIGHT (LB)
1	260.00	260.00	0.00	349.0	16.6	0.0160	21	9712.96	4194.14
2	260.00	186.91	12.50	164.8	16.4	0.0160	10	3446.23	1148.81
3	186.91	7.80	25.00	192.1	16.6	0.0160	11	940.30	492.18
TOTALS								14099.49	5835.12

NNOSE CAP DESIGN

DESIGN PRESSURE ON NNOSE CAP, PSI POSPH = 11.396
MINIMUM ALLOWABLE THICKNESS OF NNOSE CAP SKIN, IN. THINN = -0.0000
MAXIMUM ALLOWABLE TEMPERATURE OF SKIN, DEG. F. TMPMAX = 10000.0
NNOSE CAP SKIN THICKNESS REQUIRED FOR STRUCTURAL INTEGRITY, IN. TCAPST = 0.009
NNOSE CAP SKIN THICKNESS REQUIRED FOR THERMAL REASONS, IN. TCAPTH = 0.000
NNOSE CAP SKIN THICKNESS USED TO CALCULATE WEIGHT, IN. TCAP = 0.009
NNOSE CAP RADIUS, IN. RCAP = 4.303
LENGTH OF NNOSE CAP, IN. ALCAP = 1.43
NNOSE CAP SURFACE AREA, SQ. IN. SCAP = 67.18
USEFUL VOLUME OF NNOSE CAP, CU. FT. VCAP = 0.01
WEIGHT OF NNOSE CAP, LBS. WCAP = 0.06

TOTAL LENGTH, VOLUME AND WEIGHT OF FAIRING

TOTAL LENGTH OF FAIRING, IN. ALTOT = 700.37
USEFUL VOLUME OF FAIRING, CU. FEET VTOT = 14262.64
TOTAL VOLUME OF FAIRING, CU. FEET VGRSS = 15558.15
TOTAL WEIGHT OF FAIRING, LBS. WTOT = 5835.18

Figure 6A - Computer Output for Sample Problem - Design Summary

DESIGN DETAILS OF CONICAL FRUSTUMS

ANGLE DATA

TYPE	ZEC SECTION
BASE LEG H/T	HWT = 10.00
UPRIGHT LEG H/T	AWT = 20.00
OUTSTANDING LEG H/T	CAI = 6.00
FLANGE BUCKLING STRESS, PSI	FCPB = 30000.

SANDWICH FACE DATA

FACE MATERIAL

7075-T6 ALUMINUM

Figure 6B - Computer Output for Sample Problem - Design Details

FRUSTUM RAY NO.	SHELL NO. (IN)	RAY LENGTH (IN)	FACE GAUGE (IN)	RETRD. GAUGE (IN)	RING GAUGE (IN)	TOTAL FACE WT. (LB)	TOTAL CONE WT. (LB)	RING WT. (LB)	WEIGHT INDEX (LB/CH. FT)	SHELL THICKNESS (IN)	TOTAL RAY WT. (LB)	CELL WEIGHT (IN)
1-1	260.0	15.6	0.0320	0.00100	0.1900	89.47	3.14	109.65	0.4570	0.9070	234.1	1.0000
1-2	260.0	16.6	0.0200	0.00100	0.1900	54.00	4.03	109.44	0.3910	1.1450	194.3	0.5625
1-3	260.0	16.5	0.0200	0.00100	0.1900	54.00	4.03	109.44	0.3910	1.1450	194.3	0.5625
1-4	260.0	16.5	0.0250	0.00100	0.1900	57.57	3.14	109.65	0.4119	0.9000	210.1	0.8125
1-5	260.0	16.6	0.0200	0.00100	0.1900	53.98	4.48	109.34	0.3824	1.2700	195.0	0.5625
1-6	260.0	16.6	0.0200	0.00100	0.1900	53.98	5.36	109.12	0.3851	1.5200	195.4	0.5625
1-7	260.0	16.5	0.0200	0.00100	0.1900	53.95	4.92	109.23	0.3437	1.3950	195.7	0.5625
1-8	260.0	16.6	0.0200	0.00100	0.1900	54.00	4.03	109.44	0.3910	1.1450	194.3	0.5625
1-9	260.0	16.6	0.0200	0.00100	0.1900	54.00	3.59	109.55	0.3796	1.0200	193.0	0.6250
1-10	260.0	16.6	0.0400	0.00100	0.1900	108.11	3.14	109.65	0.3129	0.9150	261.6	1.0000
1-11	260.0	16.6	0.0250	0.00100	0.1900	67.44	4.92	109.23	0.4164	1.4000	213.2	0.8750
1-12	260.0	16.5	0.0200	0.00100	0.1900	53.92	3.36	109.12	0.3851	1.5200	196.4	0.6250
1-13	260.0	16.6	0.0250	0.00100	0.1900	67.53	3.59	109.55	0.3769	1.0250	210.9	0.9375
1-14	260.0	16.6	0.0200	0.00100	0.1900	54.08	2.70	109.77	0.3769	0.7700	192.2	0.6875
1-15	260.0	16.6	0.0200	0.00100	0.1900	53.98	4.48	109.34	0.3824	1.2700	195.0	0.6275
1-16	260.0	16.6	0.0320	0.00100	0.1900	86.61	1.80	109.93	0.4535	0.5320	231.3	1.0000
1-17	260.0	16.5	0.0200	0.00100	0.1900	54.11	2.25	109.87	0.3755	0.6450	191.5	0.7500
1-18	260.0	16.6	0.0200	0.00100	0.1900	54.00	4.03	109.44	0.3910	1.1450	194.3	0.7500
1-19	260.0	16.6	0.0320	0.00100	0.1900	86.36	4.48	109.34	0.4644	1.2120	236.9	1.0000
1-20	260.0	16.6	0.0200	0.00100	0.1250	54.05	3.14	47.71	0.2568	0.8950	131.0	0.8125
1-21	260.0	16.6	0.0200	0.00100	0.1250	54.38	2.76	95.50	0.2524	0.7700	131.9	0.8125
2-1	252.7	16.4	0.0250	0.00100	0.1250	57.24	5.35	46.09	0.3066	1.5250	150.2	1.0000
2-2	245.5	16.4	0.0200	0.00100	0.1250	54.26	5.19	44.75	0.2787	1.5200	128.9	0.8125
2-3	234.2	16.4	0.0200	0.00100	0.1250	50.94	1.69	43.75	0.2741	0.5200	119.5	0.8750
2-4	230.9	16.4	0.0200	0.00100	0.1250	49.11	5.09	41.86	0.3005	1.8500	123.2	0.8750
2-5	223.0	16.4	0.0250	0.00100	0.1250	59.77	2.58	40.95	0.3337	0.7750	125.5	1.0000
2-6	216.4	16.4	0.0200	0.00100	0.1250	46.31	1.92	39.64	0.3026	0.6450	104.2	0.8750
2-7	209.1	16.4	0.0200	0.00100	0.1250	44.35	1.12	38.38	0.3092	0.3950	104.3	0.9375
2-8	201.8	16.4	0.0200	0.00100	0.1250	42.24	1.80	36.93	0.3236	0.6450	101.9	0.9375
2-9	194.6	16.4	0.0200	0.00100	0.1000	41.70	1.73	28.83	0.2918	0.6450	95.4	0.9375
2-10	186.9	17.2	0.0250	0.00100	0.1000	52.71	1.75	42.63	0.3428	0.6500	97.7	1.0000
3-1	171.4	16.6	0.0200	0.00100	0.1250	41.06	1.71	31.21	0.3788	0.6450	91.8	0.6875
3-2	156.0	16.5	0.0200	0.00100	0.1000	37.17	4.69	17.63	0.3882	2.0200	78.5	0.7500
3-3	140.5	16.6	0.0200	0.00100	0.1000	33.72	1.69	16.31	0.4025	0.7700	66.8	0.8125
3-4	125.0	16.5	0.0200	0.00100	0.0900	30.42	1.01	11.79	0.4217	0.5200	56.1	0.8125
3-5	109.5	16.5	0.0200	0.00100	0.0900	26.83	1.11	10.25	0.4783	0.6450	49.7	0.8750
3-6	94.0	16.5	0.0200	0.00100	0.0800	23.25	0.96	6.94	0.5246	0.6450	41.1	0.9375
3-7	73.5	15.5	0.0200	0.00100	0.0710	19.67	0.97	4.53	0.5964	0.7700	33.6	1.0000
3-8	63.1	15.6	0.0200	0.00100	0.0630	16.07	0.92	2.83	0.7035	0.8950	26.7	1.0000
3-9	47.0	16.6	0.0200	0.00100	0.0630	12.46	0.92	2.05	0.9899	1.0200	20.7	1.0000
3-10	32.1	16.6	0.0200	0.00100	0.0500	9.11	0.15	0.91	1.1251	0.2700	13.6	1.0000
3-11	7.8	25.1	0.0320	0.00100	0.0250	11.21	0.23	0.09	2.5595	0.5320	13.6	1.0000

Figure 6C - Computer Output for Sample Problem - Design Details (Cont. 'd)

FRUSTUM -DAY NO.	NATURAL LINE LOAD (LBS/IN)	LEGARD LINE LOAD (LBS/IN)	MAX APPLIED FACE STRESS, PSI				ALLOWABLE STRESS, PSI			
			AXIAL	CIRC	SHEAR	BURST	MARKING	FACE YIELD	FACE DIMPLOING	
1-1	750.27	-1122.15	17534.	-0.	3725.	11602.	6008.	41500.	22113.	
1-2	719.27	-1090.95	27274.	-0.	4362.	18564.	6008.	41500.	27200.	
1-3	686.08	-1054.76	26494.	-0.	4362.	18564.	6008.	41500.	27300.	
1-4	656.89	-1028.57	25371.	-0.	3490.	14831.	6008.	41500.	20442.	
1-5	623.69	-997.37	24334.	-0.	4362.	18564.	6008.	41500.	27300.	
1-6	594.50	-966.18	24155.	-0.	4362.	18564.	6008.	41500.	27300.	
1-7	563.31	-934.99	23175.	-0.	4362.	18564.	6008.	41500.	27300.	
1-8	532.12	-903.80	22395.	-0.	4362.	18564.	6008.	41500.	27300.	
1-9	500.92	-872.60	21915.	-0.	4362.	18564.	6008.	41500.	22113.	
1-10	469.73	-841.41	16512.	-0.	2181.	9242.	52132.	41500.	34751.	
1-11	438.54	-810.22	16204.	-0.	3490.	14831.	6008.	41500.	17624.	
1-12	407.35	-779.02	15476.	-0.	4362.	18564.	6008.	41500.	22113.	
1-13	376.15	-747.83	14557.	-0.	3490.	14831.	6008.	41500.	15356.	
1-14	344.96	-716.64	17516.	-0.	4362.	18564.	6008.	41500.	18275.	
1-15	313.77	-685.45	17136.	-0.	4362.	18564.	6008.	41500.	14275.	
1-16	282.57	-654.25	16223.	-0.	2726.	11602.	61222.	41500.	22113.	
1-17	251.38	-623.06	15577.	-0.	4362.	18564.	6008.	41500.	15356.	
1-18	220.19	-591.87	14797.	-132.	4362.	18564.	6008.	41500.	15356.	
1-19	189.01	-560.69	8761.	-158.	2724.	11602.	6008.	41500.	22113.	
1-20	157.86	-529.54	13238.	-346.	4352.	18564.	6008.	41500.	13085.	
1-21	126.79	-498.47	12662.	-631.	4335.	18564.	6008.	41500.	13085.	
2-1	97.47	-478.18	9564.	6338.	3439.	14962.	6008.	41500.	13497.	
2-2	82.29	-463.27	11582.	7701.	4136.	18170.	6008.	41500.	13085.	
2-3	66.85	-448.44	11211.	7466.	3971.	17038.	48955.	41500.	1282.	
2-4	51.07	-433.76	10344.	7041.	3810.	17107.	6008.	41500.	11282.	
2-5	34.80	-419.25	8485.	5302.	2427.	13260.	6008.	41500.	13497.	
2-6	17.90	-404.85	10121.	6200.	3514.	16043.	6008.	41500.	11282.	
2-7	0.18	-390.53	7763.	5782.	3377.	15511.	56170.	41500.	9128.	
2-8	-18.46	-376.22	9405.	5378.	3247.	14974.	6008.	41500.	9128.	
2-9	-38.46	-361.93	9048.	4590.	2134.	14448.	6008.	41500.	9128.	
2-10	-60.52	-347.52	6550.	3072.	2441.	11120.	48955.	41500.	13497.	
3-1	-93.31	-358.01	8550.	13722.	3034.	14008.	6008.	41500.	18275.	
3-2	-85.90	-327.83	8196.	14420.	2778.	12788.	6008.	41500.	15356.	
3-3	-78.51	-297.62	7441.	12118.	2522.	11569.	6008.	41500.	13085.	
3-4	-71.12	-267.38	6884.	11816.	2254.	10349.	48955.	41500.	13085.	
3-5	-63.73	-237.09	5927.	10513.	2006.	9129.	6008.	41500.	11282.	
3-6	-56.33	-206.77	5169.	9211.	1747.	7910.	6008.	41500.	9128.	
3-7	-48.66	-176.41	4416.	7909.	1486.	6650.	6008.	41500.	4538.	
3-8	-41.26	-146.09	3652.	6607.	1222.	5473.	6008.	41500.	8638.	
3-9	-33.48	-116.12	2803.	5224.	960.	4251.	6008.	41500.	6038.	
3-10	-25.70	-86.81	2170.	3976.	707.	3031.	67439.	41500.	5538.	
3-11	-17.96	-56.34	512.	1605.	289.	877.	61222.	41500.	22113.	

Figure 6D - Computer Output for Sample Problem - Loads and Stresses

DETAILED LOADS DATA

POSITION -RAY NO.	RAY BASE DIST. (IN)	RAY TOP DIST. (IN)	RAY LENGTH (IN)	DISTANCE FROM BASE (IN)	DES. PRES. WINDNO (PSF)	DES. PRES. LEEWARD (PSF)	AXIAL LOAD (LBS)	SPAN L-CL (LBS)	ACROSS MEMBER (L-CL)
1-1	260.0	260.0	16.6	0.0	-0.00	-0.00	108426.4	71251.7	33508090.0
1-2	260.0	260.0	16.6	16.6	-0.00	-0.00	108426.4	71251.7	36125151.0
1-3	260.0	260.0	16.6	33.2	-0.00	-0.00	108426.4	71251.7	33142100.8
1-4	260.0	260.0	16.6	49.8	-0.00	-0.00	108426.4	71251.7	31959292.1
1-5	260.0	260.0	16.6	66.4	-0.00	-0.00	108426.4	71251.7	30776318.3
1-6	260.0	260.0	16.6	83.0	-0.00	-0.00	108426.4	71251.7	29593374.2
1-7	260.0	260.0	16.6	99.6	-0.00	-0.00	108426.4	71251.7	28410430.3
1-8	260.0	260.0	16.6	116.2	-0.00	-0.00	108426.4	71251.7	27227435.2
1-9	260.0	260.0	16.6	132.8	-0.00	-0.00	108426.4	71251.7	26044341.0
1-10	260.0	260.0	16.6	149.4	-0.00	-0.00	108426.4	71251.7	24861397.0
1-11	260.0	260.0	16.6	166.0	-0.00	-0.00	108426.4	71251.7	23678453.3
1-12	260.0	260.0	16.6	182.6	-0.00	-0.00	108426.4	71251.7	22495708.0
1-13	260.0	260.0	16.6	199.2	-0.00	-0.00	108426.4	71251.7	21312764.3
1-14	260.0	260.0	16.6	215.8	-0.00	-0.00	108426.4	71251.7	20129820.2
1-15	260.0	260.0	16.6	232.4	-0.00	-0.00	108426.4	71251.7	18946876.0
1-16	260.0	260.0	16.6	249.0	-0.00	-0.00	108426.4	71251.7	17763931.8
1-17	260.0	260.0	16.6	265.6	-0.00	-0.00	108426.4	71251.7	16580987.6
1-18	260.0	260.0	16.6	282.2	-0.00	-0.00	108426.4	71251.7	15398096.2
1-19	260.0	260.0	16.6	298.8	-0.00	-0.00	108426.4	71251.7	14215584.3
1-20	260.0	260.0	16.6	315.4	-0.00	-0.00	108426.4	71251.7	13034303.1
1-21	260.0	260.0	16.6	332.0	-0.00	-0.00	108426.4	71251.7	11856044.4
2-1	260.0	260.0	16.6	348.6	-0.00	-0.00	108426.4	71251.7	10677785.2
2-2	260.0	260.0	16.6	365.2	-0.00	-0.00	108426.4	71251.7	9501899.6
2-3	260.0	260.0	16.6	381.8	-0.00	-0.00	108426.4	71251.7	8326241.8
2-4	260.0	260.0	16.6	398.4	-0.00	-0.00	108426.4	71251.7	7150669.2
2-5	260.0	260.0	16.6	415.0	-0.00	-0.00	108426.4	71251.7	5975113.9
2-6	260.0	260.0	16.6	431.6	-0.00	-0.00	108426.4	71251.7	4800078.1
2-7	260.0	260.0	16.6	448.2	-0.00	-0.00	108426.4	71251.7	3625424.8
2-8	260.0	260.0	16.6	464.8	-0.00	-0.00	108426.4	71251.7	2450880.4
2-9	260.0	260.0	16.6	481.4	-0.00	-0.00	108426.4	71251.7	12763307.9
2-10	260.0	260.0	16.6	498.0	-0.00	-0.00	108426.4	71251.7	11018312.8
2-11	260.0	260.0	16.6	514.6	-0.00	-0.00	108426.4	71251.7	9273096.1
2-12	260.0	260.0	16.6	531.2	-0.00	-0.00	108426.4	71251.7	7527528.3
2-13	260.0	260.0	16.6	547.8	-0.00	-0.00	108426.4	71251.7	5781955.4
2-14	260.0	260.0	16.6	564.4	-0.00	-0.00	108426.4	71251.7	4036402.7
2-15	260.0	260.0	16.6	581.0	-0.00	-0.00	108426.4	71251.7	2290877.4
2-16	260.0	260.0	16.6	597.6	-0.00	-0.00	108426.4	71251.7	585607.3
2-17	260.0	260.0	16.6	614.2	-0.00	-0.00	108426.4	71251.7	236638.7
2-18	260.0	260.0	16.6	630.8	-0.00	-0.00	108426.4	71251.7	104339.0
2-19	260.0	260.0	16.6	647.4	-0.00	-0.00	108426.4	71251.7	33546.4
2-20	260.0	260.0	16.6	664.0	-0.00	-0.00	108426.4	71251.7	35108.2
2-21	260.0	260.0	16.6	680.6	-0.00	-0.00	108426.4	71251.7	10579.3
2-22	260.0	260.0	16.6	697.2	-0.00	-0.00	108426.4	71251.7	158.2
2-23	260.0	260.0	16.6	713.8	-0.00	-0.00	108426.4	71251.7	

TOTAL PT. 7.0

Figure 6E - Computer Output for Sample Problem - Detailed Loads

2.0 DESCRIPTION OF SUBROUTINES

2.1 Subroutine THERML (Thermal Computations)

This subroutine computes the minimum thickness of skin required to limit the skin temperature to the specified maximum. Minimum thicknesses are computed for both the nose cap and top frustum. The analytical basis for the computations performed in Subroutine THERML is presented in Appendix J. The analysis is based on a nominal two-stage Saturn V ascent trajectory to a 100 nautical mile circular orbit.

The equations and stored coefficients in Subroutine THERML were developed by means of a multiple regression analysis of a large amount of analytical data generated for the trajectory mentioned above. Details of the multiple regression analysis are also presented in Appendix J.

The parameters required from the main program for computations in Subroutine THERML are RCAP, THETA, TMPMAX and MAT. Returned to the main program are TCONTH and TCAPTH.

2.2 Subroutine TNO SST (Nose Cap Structure)

Subroutine TNO SST calculates the nose cap skin thickness required to withstand the pressure differential, PDSPH, at the stagnation point. The method used to calculate PDSPH and the structural analysis of a spherical nose cap are described in Appendix H.

Parameters required by Subroutine TNO SST are PDSPH, E, RCAP and TMINN. Returned to the main program is TCAPST.

2.3 Subroutine AERO (Pressure Coefficients)

When a pressure profile is not input to the program the pressure coefficient data for the profile is computed in Subroutine AERO. The analytical basis for the computations performed in this subroutine is presented in Appendix C. Because of the assumptions made in computing this data, the pressure coefficient at zero angle-of-attack and the change in pressure coefficient due to angle-of-attack are uniform over the length of each frustum. The pressure profile is then constructed by assigning the pressure coefficient data for each frustum to the beginning and end points of each frustum. Double points occur at the intersection of two frustums.

Parameters required to make the computations in Subroutine AERO are NF, THTA (NF), AMACH and ALPHA. Returned to the main program are CPOO and CPAA.

A sinusoidal pressure distribution in the circumferential direction is used in computing CPAA.

2.4 Subroutine LOAD (Bending Moment, Axial Force and Shear Force)

Using the pressure profile data, Subroutine LOAD computes the bending moment, axial force and shear force at the base of each bay. Bending moment and axial force are then converted to line loads (force per unit length of circumference) for both the windward and leeward sides of the bay, and the factor of safety is applied to these line loads.

When Subroutine LOAD is called upon for loads on the first bay, the bending moment, axial force and shear force are computed at the base of the fairing. As design of the fairing moves from the base to the nose cap, increments of the loads contributed by the pressure profile between the previous bay location and the new bay location are subtracted from the previous totals. Derivation of the equations used in this subroutine appear in Appendix D.

Information required for the computations in Subroutine LOAD is

1. Pressure profile data:

LTMAX

CPO (LT) LT = 1, LTMAX

CPA (LT) LT = 1, LTMAX

XOD (LT) LT = 1, LTMAX

2. Fairing geometry:

NFMAX

DBAS

ALF (NF) NF = 1, NFMAX

THTA (NF) NF = 1, NFMAX

3. Aerodynamic data:

AMACH

QBAR

DELTAP

ALPHA

4. Miscellaneous:

DSUBB

C2

FS

LFLG

LTRIG

Parameters computed by Subroutine LOAD are ANFIMX, ANFIMN, FSUBZ and LTUNCT (NF) for NF = 1 to NFMAX.

2.5 Subroutine PRESUR (Lateral Pressure)

Using pressure profile data, Subroutine PRESUR computes the pressure differential across the skin along the length of the bay. The maximum differential occurring along the bay length is determined for both the windward and leeward sides, multiplied by the factor of safety and returned to the main program. The equations used in this subroutine are developed in Appendix E.

Parameters required for computations performed in Subroutine PRESUR are

1. Pressure profile data

LTMAX

CPO (LT) LT = 1, LTMAX

CPA (LT) LT = 1, LTMAX

XOD (LT) LT = 1, LTMAX

2. Other parameters: LPFL, C2, DBAS, DSUBB, ALB, QBAR, NFMAX, NF, DELTAP, FS and LTUNCT (NF) for NF = 1 to NFMAX.

Parameters computed in Subroutine PRESUR are PDESMN, PDESMX, ALN and ALX

2.6 Subroutine PROPTY (Material Properties)

Properties of several commonly used materials are stored in this subroutine.

The material for which properties are desired is indicated by the input parameter MAT.

Output parameters are the material properties E, AMU, RHO, and TMPMAX.

2.7 Subroutine DIAM (Local Diameter)

When given the parameters describing the external geometry of the fairing and the distance from the tip of the nose cap in calibers this subroutine computes the local fairing diameter.

Parameters required by this subroutine are:

ALTOT

ALF (NF), NF = 1, NFMAX

THTA (NF), NF = 1, NFMAX

DMN (NF), NF = 1, NFMAX

XN

NFMAX

The local diameter, DLOC, is computed by this subroutine.

2.8 Subroutine DLOD (Incremental Loads)

When given the geometry and pressure coefficient data for an incremental length of the fairing this subroutine computes the contribution by this increment to the total bending moment, axial load and shear load.

Input parameters are XOD1, XOD2, D1, D2, CP01, CP02, CPA1, CPA2, A3, A4, and DP.

Parameters computed in Subroutine DLOD are FSBZ, BND, and AXLOD.

2.9 Subroutine RING (Ring Strength and Stiffness)

When given the cross-sectional shape of the ring and the skin gauge from which it is to be fabricated, this subroutine computes the moment of inertia of the ring with and without the effective skin, its cross-sectional area, eccentricity and torsion constant. This subroutine also calls Subroutine RSTRES which computes the local stress level in the ring.

Input parameters required by RING are E, IKIND, AOT, BOT, COT, WSEF, J, K, PI, and T(J) when J = 1, 30.

Output parameters are AIRING, AISTT, AST, A, Z, ECC and TCONST.

2.10 Subroutine RSTRES (Local Stress Level in Ring)

When designing a stiffener ring it is necessary to check for local instability in the ring flange or web (see Appendix G). Subroutine RSTRES computes the local stress level in the ring. This computed stress level is then compared to an input flange buckling stress level, and, if necessary, a greater web thickness is assigned to the ring.

Input parameters for Subroutine RSTRES are A, T(J), Z, PDESMX, DSUBB, CTH, AL(J), AST, and AISTT.

The output parameter is FRING.

2.11 Subroutine IREQ (Required Moment of Inertia)

This subroutine computes the stiffening ring moment of inertia requirements to prevent general instability of the structure. The methods used to compute this required moment of inertia are described in Appendix G.

Input parameters are THTA (NF), C6, AL(J), DSUBB, C2, PDESMX, T(J), PI, ANFIMX, E and ALCONE.

The output parameter is AIREQ.

2.12 Subroutine INSTBL (Panel Stability)

When given details of a tentative sandwich design for a bay, along with applied crushing or bursting pressure, INSTBL computes the line load at which the bay will fail due to overall instability. The analysis was prepared by B. O. Almroth of LMSC's Solid Mechanics Laboratory, and is described in Appendix K.

Subroutine INSTBL requires a large amount of tabular information which, for reasons of convenience, is read into the program as regular input data. This data is physically located immediately after the program deck. Nose fairing data cards for a number of nose fairing designs can be stacked in the usual manner behind these cards.

Input parameters required by subroutine INSTBL are EFACE, G1, G2, AMU, T1, T2, APRC, H, KY, PXL, and KG. Output from the subroutine is CRG, the initial line load, which is compared to the applied line load at the position (windward or leeward) then under investigation.

3.0 INPUT FORMAT

When the pressure profile and nose cap lift and drag data are to be computed in the program, the following three types of input data cards are the only types required.

Types 1, 2, 3, and 4 Parameters which apply to the entire fairing.

Type 5 Parameters which apply to individual frustum
(one card per frustum).

When lift and drag data for the nose cap are to be input, or when a pressure profile is to be input, the following additional card type is required.

Type 6 Lift and drag data for the nose cap.

When pressure profile data is to be input, two additional card types are required.

Type 7 Pressure profile data points (one card per
data point.)

Type 8 Card indicating end of pressure profile data.

The detailed format for these cards is as follows:

Type 1: Format (5F12.8, 2I6)

Data:	DBAS	- Base diameter of fairing, in.
	QBAR	- Dynamic pressure at design point in the trajectory, lbs/sq. ft.
	AMACH	- Mach number at design point in the trajectory.
	ALPHA	- Angle-of-attack at design point in the tra- jectory, degrees.
	KEY	- A integer indicating the type of output desired. The code is as follows:
		(0) Design summary only. (See Figure 6A).
		(1) Design summary plus design details (Figures 6B and 6C.)
		(2) Design summary plus design details plus load details (Figures 6C, 6D and 6E.)

Type 2: Format (2I6, 4F12.8, 1I2)

Data:	MAT	- An integer indicating the material to be used. The code is as follows:
		(1) aluminum
		(2) magnesium

- (3) titanium
 - (4) stainless steel
 - (5) Lockalloy (a Be-Al alloy)
- NFMAX - The number of conical frustums in the fairing.
- TMP - The maximum allowable skin temperature for the nose cap and top frustum, °F. (If no value is input, the value stored with the material properties is used. If a value equal to or greater than 10,000 is input, no thermal constraint is imposed on the skin thickness.)
- TMINN - The minimum skin gauge to be used in nose cap design, in.
- DELTAP - Difference between internal and free-stream pressure, psi
- FS - Factor of safety. (If no value is input, a factor of safety of 1.4 is used.)
- LPRES - An integer indicating whether nose cap lift and drag data and/or pressure profile data will be input. If pressure profile data is input, LPRES also indicates the type of lift data to be input. The code is as follows:
- (-1) Lift and drag data for the nose cap is input, but no pressure profile data is input.
 - (0) No nose cap or pressure profile data is input.
 - (1) Nose cap data and pressure profile data with CPA (LT) as defined in Figure 5 and Appendix B is input on Card type 5.
 - (2) Same as (1) except that CPA (LT) per radian angle-of-attack is input for CPA (LT) on Card type 5.
 - (3) Same as (1) except that

$$\frac{\partial}{\partial X} \left(\frac{\partial C_N}{\partial \alpha} \right) \text{ (See Section 1.4) is input for CPA (LT) on Card type 5.}$$

Type 3: Stringer Data (I5, 6E12.8)

Data: IKIND - Type of cross-section

1 = angle

2 = zee

3 = hat

AOT - Web height-to-thickness ratio (see Figure 3)

BOT - Flange width-to-thickness ratio (see Figure 3)

COT - Hat section flange width-to-thickness ratio (see Figure 3)

FCFB - Ring outstanding flange buckling level, psi

Type 4: Face Material (2I6, 4F12.8, 1I2)

Data: MATF - Type of material

1 = 2024-T4 Alclad Aluminum

2 = 2024-T4 Aluminum

3 = 7075-T6 Aluminum

FTA - Face tensile allowable stress

FCA - Face compressive allowable stress

FSA - Face shear allowable stress

EFACE - Modulus of elasticity of face material

One Type 5 card is required for each frustum.

Type 5: (5F12.8, 2I6)

Data: ALF (NF) - Ratio of top diameter of frustum to base diameter of fairing or length of the frustum in inches. If the number is equal to or greater than 1, it will be treated as frustum length. For conical sections, either the diameter ratio or length can be used. For cylindrical sections, only length can be input.

THTA (NF) - Frustum half angle, degrees.

ELMIN (NF)	- Bay length to be used in designing frustum, in.
TMNC (NF)	- Minimum face thickness to be used in designing frustum, in.
TRBMIN (NP)	- Minimum ribbon thickness to be used in designing frustum, in.
KTRSTD	- An integer indicating whether standard ribbon gauge material is to be used 0 - Use non-standard gauges 1 - Use standard gauges
KTFSTD	- An integer indicating whether standard gauge material is to be used for sandwich faces. 0 - Use non-standard gauges 1 - Use standard gauges

The next card type is required only when nose cap lift and drag data or a pressure profile is read in (PRES = -1, 1, 2 or 3 in Card Type 2). If a blank card is inserted for Card Type 6 when LPRES = 1, 2 or 3, the program will compute CDCAP, CNCAP and XBCAP.

Type 6: Format (3F12.8)

Data:	CDCAP	- Spherical nose cap drag coefficient with nose cap base area as a reference area.
	CNCAP	- Normal force coefficient per radian angle-of-attack for the nose. Reference area is nose cap base area.
	XBCAP	- Distance from base of nose cap to center of pressure for the nose cap, in.

The next two card types are required only when pressure profile data is input (LPRES = 1, 2 or 3). (See Section 1.4 and Figure 4.)

Type 7: Format (3F12.8)

Data:	CPO (LT)	- Zero angle-of-attack pressure coefficient at station LT.
	CPA (LT)	- Lift parameter at station LT. See LPRES on Card Type 2 and Section 1.4 for options which are available.
	XOD (LT)	- Local to base diameter ratio at station LT.

(One card is required for each data point, starting with the first point at the junction of the nose cap and top frustum.)

Type 8: Format (71X, 11)

Data: LSTOP = 1 - This signals the computer that the last pressure profile data point has been read in.

The set of data cards described above will design one fairing. A number of fairings can be designed with one computer run by placing several sets of data cards behind the program deck.

4.0 SAMPLE PROBLEM

The following sample problem illustrates the input format of the program. Input data is as follows:

Fairing geometry:

DBAS	= 260.0 inches
NFMAX	= 3
ALF (1)	= 349.0 inches
ALF (2)	= 0.72
ALF (3)	= 0.03
THTA (1)	= 0.00 degrees
THTA (2)	= 12.5 degrees
THTA (3)	= 25.0 degrees

Design specifications:

ELMIN (1)	= 16.6 inches
ELMIN (2)	= 16.4 inches
ELMIN (3)	= 16.6 inches
TMNC (1)	= 0.016 inches
TMNC (2)	= 0.016 inches
TMNC (3)	= 0.016 inches
KTFSTD	= 1 (use standard gauges for faces)
TRBMIN (1)	= 0.001 inches
TRBMIN (2)	= 0.001 inches
TRBMIN (3)	= 0.001 inches
KTRSTD	= 1 (use standard gauges for ribbon)
TMINN	= 0.0 inches
TMP	= 10000.0°F

One ring shape is specified for all three frustums.

IKIND	= 2
AOT	= 20.0
BOT	= 10.0
COT	= 0.0
FCFB	= 30000.0 psi

Material:

MAT = 1 (aluminum)

Face Material:

MATF = 1 (7075-T6 Aluminum)
FTA = 58000. psi
FCA = 41500. psi
FSA = 27000. psi
EFACE = 10300000. psi

Aerodynamic data:

AMACH = 1.5
QBAR = 765.0 lbs/sq ft
ALPHA = 8.5 degrees
DELTAP = 0. psi

Factor of Safety:

FS = 1.4 (It is not necessary to input this value,
since 1.4 is the value which the program
uses when no value is indicated.)

Program Controls:

LPRES = 1
KEY = 2
DELTAL = 0.1 inches

Nose cap lift and drag data:

A blank card is inserted in the deck in place of Card Type 4,
causing the program to compute this data.

Pressure profile data:

Data for the pressure profile is taken from Figure 4 and listed
in Figure 8 under Card Type 7.

This input data is arranged in key-punch format in Figure 7. The
computer output for this problem is shown in Figures 6A, 6B, 6C and 6D.

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APPENDIX A
MAIN PROGRAM LISTING

```

$JOB          LMSC-C53/ADAMS SAB ,460040,00,12,140CEP
$*            NOSE FAIRING OPTIMIZATION PROGRAM
$EXECUTE      IBJOB
$IBJOB LMSC   MAP
$IBFTC C53S   DECK
C             STRUCTURAL OPTIMIZATION AND DESIGN OF MULTI-FRUSTUM NOSE
C             FAIRINGS USING RING AND SANDWICH CONSTRUCTION.
               DIMENSION WSEG(400), VSEG(400), T(30), D(400), ALOPT(400) ,
1 TRING(400), WRING(400), WIDEX(400), ENFIMX(400), ENFIMN(400),
2 FFWR(400), FFDMP(400), ELMIN(400), TMNC(400), IMX(400), CELSZ(400),
3 FZ(400), TRBN (400), WFACE(400), WTOTFC(400), TRBMIN(400), TRBSTD( 8),
4 WTCORE(400), TFACE(400), FFX(400), FFC(400), FFS(400), FFB(400),
5 WSPLCE(400), TCORE(400)
               INTEGER CASE
               COMMON ANFIMN, ANFIMX, DSUBB, PDESMN, PDESMX, E
               COMMON /THRML/ TMPMAX, MAT, TCONTH, TCAPTH, THETA
               COMMON/AERPRS/CPAA, CPOO, LPFL
               COMMON/INSTB/EFACE, G1, G2, AMU, T1, T2, APRC, H , CRG, KY, P, KG
               COMMON/LOADS/CDCAP, CNCAP, XBCAP, FSUBZ, AXLDCP, FSBZCP, BNDCAP, LTRIG,
1  ALCAP
               COMMON/KLSS/XL
               COMMON/PRESR/CPA(121), CPO(121), XOD(121), DELTAP, FS, LTMAX, SUMAL,
1  QBAR, ALTOT
               COMMON /CHKLD/ C1, C3, C4, ALB, DELTAS, CHK, RAXMIN, RAXMAX, RPMIN, RPMAX,
1  CHKWND, CHKLEE, C2
               COMMON/SFCTN/ GBAR, F
               COMMON/CONFG/THTA(10), ALF(10), NFMAX, DBAS, DMN(10), ICONF
               COMMON/AERLOD/ALPHA, AMACH, NF
               COMMON/IRG/ TTM, PI, ALCONE, C6, AIREQ
               COMMON/RNG/ IKIND, AOT, BOT, COT, TSID(30), K, AIRING, DD
               COMMON /RSTRSS/ AA, TF, Z, CTH, AST, AISTT, FRING
               COMMON/TNOS/TMINN, PDSPPH, TCAPST, RCAP
C
               CASE      = 0
               PI        = 3.1415927
C             READ PARAMETERS WHICH APPLY TO ENTIRE FAIRING.
10 CONTINUE
               READ (5,131) DBAS, DELTAL, QBAR, AMACH, ALPHA, KEY
               READ (5,132) MAT, NFMAX, TMP, TMINN, DELTAP, FS, LPRES

```

```

C      READ(5,989) IKIND ,AOT,BOT,COT,FCFB
C      IF (NFMX .GT. 10) GO TO 98
C      READ SANDWICH FACE MATERIAL DATA. DO NOT USE ALLOWABLE
C      STRESSES IN EXCESS OF PROPORTIONAL LIMIT, SINCE BUCKLING
C      ANALYSES ASSUME ELASTIC BEHAVIOR.
C      READ (5,990) MATF, FTA, FCA, FSA, EFACE
C      ALTOT = 0.
C      DMX = DBAS
C      DO 30 NF = 1,NFMX
C      READ PARAMETERS WHICH DESCRIBE INDIVIDUAL FRUSTUMS.
C      READ(5,131) ALF(NF),THTA(NF),ELMIN(NF),TMNC(NF),TRBMIN(NF),KTRSTD,SWD0050
C      1KTFSTD
C      THETA = 0.0174532925*THTA(NF)
C      IF (ALF(NF) .LT. 1.) GO TO 20
C      DMN(NF) = DMX - 2.*ALF(NF)*SIN(THETA)/COS(THETA)
C      GO TO 25
C      20 CONTINUE
C      DMN(NF) = DBAS*ALF(NF)
C      ALF(NF) = (DMX-DMN(NF))*COS(THETA)*.5/SIN(THETA)
C      25 CONTINUE
C      DMX = DMN(NF)
C      ALTOT = ALTOT + ALF(NF)
C      30 CONTINUE
C      DMX = DBAS
C      CHECK TO DETERMINE IF PRESSURE PROFILE DATA IS TO BE
C      READ IN.
C      IF (LPRES .EQ. 0) GO TO 45
C      READ NOSE CAP LIFT AND DRAG DATA IF INPUT INSTEAD OF
C      COMPUTED.
C      READ (5,131) CDCAP,CNCAP,XBCAP
C      IF (LPRES .LT. 0) GO TO 50
C      READ PRESSURE PROFILE DATA.
C      DO 35 LT = 1,101
C      READ (5,365) CPO(LT),CPA(LT),XOD(LT),LSTOP
C      IF (LSTOP .GT. 0) GO TO 40
C      35 CONTINUE

```

```

40 CONTINUE
   LTMAX = LT - 1
   GO TO 50
45 CONTINUE
   CDCAP = 0.
50 CONTINUE
   CASE = CASE + 1
      WRITE OUT HEADINGS FOR DATA REPORT.
      WRITE (6,501)
      WRITE (6,550)
      WRITE (6,555) CASE
      IF A FACTOR-OF-SAFETY WAS INPUT, THE INPUT VALUE IS
      USED. OTHERWISE THE FACTOR-OF-SAFETY IS 1.4.
      IF (FS .LT. 1.) FS = 1.4
      RADIUS OF SPHERICAL NOSE CAP IS COMPUTED.
      THETA = 0.0174532925 * THTA(NFMAX)
      RCAP = 0.5*DMN(NFMAX)/COS(THETA)
      ALCAP = RCAP*(1.-SIN(THETA))
      ALTOT = ALTOT + ALCAP
      IF NOSE CAP RADIUS IS ZERO, A NOTE IS WRITTEN OUT, AND
      MAXIMUM ALLOWABLE TEMPERATURE IS SET EQUAL TO 10000. THIS
      CAUSES THE PROGRAM TO BY-PASS THE THERMAL EQUATIONS, WHICH
      ARE NOT VALID FOR A NOSE CAP RADIUS OF ZERO.
      IF (RCAP .GT. 0.) GO TO 55
      WRITE (6,178)
      TMP = 10000.
55 CONTINUE
      PROPERTIES ARE LOOKED UP FOR THE INPUT MATERIAL. IF
      NO MAXIMUM TEMPERATURE WAS SPECIFIED, A NOMINAL VALUE WILL
      BE CHOSEN AND USED.
      CALL PROPTY (MAT, E, AMU, RHO, TMPMAX)
      CB = SQRT (1. - AMU**2)
      C4 = (PI**2 * E) / (12. * CB)
      IF MAXIMUM ALLOWABLE TEMPERATURE IS EQUAL TO OR GREATER
      THAN 10000., EITHER FROM INPUT DATA OR BECAUSE NOSE CAP
      RADIUS EQUALS ZERO, THE REQUIRED THERMAL THICKNESS OF BOTH
      THE NOSE CAP AND TOP FRUSTUM ARE SET EQUAL TO ZERO.
      IF (TMP .LT. 10000.) GO TO 60
      TMPMAX = TMP

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```

C          TCONTH = 0.0
C          TCAPTH = 0.0
C          GO TO 65
C          IF THE INPUT VALUE FOR MAXIMUM ALLOWABLE TEMPERATURE
C          IF EQUAL TO 0., THE STORED VALUE IS USED. IF THE INPUT VALUE
C          IF GREATER THAN 0. BUT LESS THAN 1000., THE INPUT VALUE IS
C          USED FOR MAXIMUM ALLOWABLE TEMPERATURE.
C          60 IF (TMP .GT. 0.) TMPMAX = TMP
C              THE SKIN THICKNESSES REQUIRED TO KEEP SKIN TEMPERATURE
C              BELOW THE MAXIMUM SPECIFIED ARE COMPUTED FOR BOTH THE NOSE
C              CAP (TCAPTH) AND THE TOP FRUSTUM (TCONTH) IN SUBROUTINE
C              THERML.
C          CALL THERML
C          65 CONTINUE
C          DATA T/ 0.001, 0.020, 0.040, 0.060, 0.080, 0.100, 0.120, 0.140,
C          1 0.160, 0.180, 0.200, 0.220, 0.240, 0.260, 0.280, 0.300, 0.320,
C          2 0.340, 0.360, 0.380, 0.400, 0.500, 0.600, 0.700, 0.800, 0.900,
C          3 1.000, 1.250, 1.500, 2.000 /
C          STANDARD RIBBON THICKNESSES
C          DATA TRBSTD/ .0007, .001, .0015, .002, .0025, .003, .004, .005 /
C          AMBIENT PRESSURE AT DESIGN CONDITIONS AND DESIGN PRESSURES
C          ON THE NOSE CAP ARE COMPUTED.
C          PSTAT = QBAR / ( 100.8 * AMACH**2)
C          PDSPH = FS*(PSTAT*((166.92158*(AMACH)**7.)/((7.*(AMACH)**2)
C          1 -1.))**2.5)-PSTAT - DELTAP)
C          AERODYNAMIC DATA ARE WRITTEN OUT.
C          WRITE (6,307)
C          WRITE (6,72) QBAR
C          WRITE (6,73) AMACH
C          WRITE (6,74) ALPHA
C          WRITE (6,163) DELTAP
C          WRITE (6,162) PSTAT
C          DESIGN CONSTRAINTS FOR FRUSTUM SECTION ARE WRITTEN OUT.
C          WRITE (6,308)
C          WRITE (6,96) TMPMAX
C          WRITE (6,164) TCONTH
C

```

```

C          HEADINGS FOR SUMMARY REPORT ARE WRITTEN OUT.
      WRITE (6,309)
      WRITE (6,310)
      WRITE (6,311)
      WRITE (6,312)
      WRITE (6,607)
C          CONVERT ANGLE OF ATTACK FROM DEGREES TO RADIANS.
      ALPHA
      = 0.0174532925 * ALPHA
      CHECK TO DETERMINE IF PRESSURE PROFILE DATA HAS BEEN
      INPUT.
C          IF (LPRES .GT. 0) GO TO 398
      C          SINCE PRESSURE DATA WAS NOT INPUT, PRESSURE PROFILE
      C          DATA IS COMPUTED IN SUBROUTINE AERO.
      ALPR = ALTOT
      LTMAX = 2*NFMAX
      LT = LTMAX + 1
      DO 70 NF = 1, NFMAX
      CALL AERO
      LT = LT - 1
      CPO(LT) = CPOO
      CPA(LT) = CPAA
      XOD(LT) = ALPR/DBAS
      ALPR = ALPR - ALF(NF)
      LT = LT - 1
      CPO(LT) = CPOO
      CPA(LT) = CPAA
      XOD(LT) = ALPR/DBAS
      70 CONTINUE
      GO TO 437
      398 CONTINUE
C          PRESSURE PROFILE DATA WAS INPUT. THE TYPE OF DATA IS
      C          INDICATED BY LPRES.
      GO TO (437,438,440) ,LPRES
      438 CONTINUE
C          SLOPE OF THE NORMAL FORCE COEFFICIENT CURVE HAS BEEN
      C          INPUT. THESE VALUES ARE CONVERTED TO THE MAXIMUM CHANGE
      C          CAUSED IN THE LOCAL PRESSURE COEFFICIENT BY THE SPECIFIED
      C          ANGLE-OF-ATTACK.

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```

SWD0193
SWD0194
SWD0195
SWD0196
SWD0197
SWD0198
SWD0199
SWD0200
SWD0201
SWD0202
SWD0203
SWD0204
SWD0205
SWD0206
SWD0207
SWD0208
SWD0209
SWD0210
SWD0211
SWD0212
SWD0213
SWD0214
SWD0215
SWD0216
SWD0217
SWD0218
SWD0219
SWD0220
SWD0221
SWD0222
SWD0223
SWD0224
SWD0225
SWD0226
SWD0227
SWD0228
SWD0229
SWD0230
SWD0231

DO 439 LT = 1,LTMAX
  XN = XOD(LT)*DBAS
  CALL DIAM (XN,DLOC)
  DOD = DLOC/DBAS
  439 CPA(LT) = 0.5*ALPHA*CPA(LT)/DOD
  GO TO 437
440 CONTINUE
C
C   THE CHANGE IN SURFACE PRESSURE COEFFICIENT PER RADIAN
C   ANGLE-OF-ATTACK HAS BEEN INPUT. THESE VALUES ARE CONVERTED
C   TO THE MAXIMUM CHANGE CAUSED IN THE LOCAL PRESSURE
C   COEFFICIENT BY THE SPECIFIED ANGLE-OF-ATTACK.
  DO 441 LT = 1,LTMAX
    441 CPA(LT) = ALPHA*CPA(LT)
    437 CONTINUE
C
C   AT THIS POINT THE PRESSURE PROFILE DATA AT SPECIFIED
C   DIAMETER RATIOS IS IN THE FORM OF SURFACE PRESSURE
C   COEFFICIENT AT ZERO ANGLE-OF-ATTACK AND THE MAXIMUM CHANGE
C   IN PRESSURE COEFFICIENT DUE TO ANGLE-OF-ATTACK.
  PARAMETERS USED IN DESIGN OF THE ENTIRE FAIRING ARE
  INITIALIZED.
    SUMAL = 0.
    VGROSS = 0.0
    I = 0
    WTOT = 0.
    VTOT = 0.
    DBASE = DBAS
    LPFL = 1
    LTRIG = -1
  DO-LOOP WHICH DESIGNS THE FRUSTUMS IS INITIALIZED.
C
C
DO 129 NF = 1,NFMAX
  CONSTANTS ARE EVALUATED AND PARAMETERS ARE INITIALIZED
  FOR THE FRUSTUM BEING DESIGNED.
    THETA = 0.0174532925*THETA(NF)
    CTH = COS(THETA)
    STH = SIN(THETA)
    C1 = CB / (0.3 * CTH)
    C2 = 2.*STH/CTH

```

```

C3      = 0.3/CTH
C5      = PI / CTH
C6      = (0.25/CTH**2)*(DBASE/(5.51*E*C2))**1.33333
C7      = STH/CTH
DMIN    = DMN(NF)
NBAY    = 0
ALCONE  = ALF(NF)
ALB     = ELMIN(NF)
TMINC   = TMNC(NF)
WCONE   = 0.
SLOPT   = 0.
VUSE    = 0.
C
C      IDENTIFY CONE BASE DIAMETER AS BASE DIAMETER OF BOTTOM
C      SEGMENT.
C      DSUBB = DBASE
C      IF THE TOP FRUSTUM IS BEING DESIGNED, REQUIRED THERMAL
C      THICKNESS AND MINIMUM ALLOWABLE THICKNESS ARE COMPARED. THE
C      GREATER OF THE TWO IS USED AS MINIMUM ALLOWABLE THICKNESS
C      FOR THE TOP FRUSTUM.
C      IF (NFMAX.GT. NF) GO TO 101
C      IF (TCONTH.GT. TMINC) TMINC = TCONTH
C      DESIGN OF THE INDIVIDUAL BAY BEGINS AT THIS POINT.
C      101 CONTINUE
C      J      = 2
C      I      = 1 + 1
C      IF (I.GT. 400) GO TO 98
C      THE REMAINING LENGTH OF THE FRUSTUM IS COMPUTED.
C      ALMX2   = ALF(NF)-SLOPT
C      IF ((2.*ALB).GT. ALMX2) ALB = ALMX2
C      LINE LOADS AND SHEAR LOADS AT THE BASE OF THE BAY ARE
C      COMPUTED IN SUBROUTINE LOAD. THEN REINDETIFIED AND STORED
C      FOR OUTPUT PURPOSES.
C      CALL LOAD
C      ENFIMN(I) = ANFIMN
C      ENFIMX(I) = ANFIMX
C      FZ(I)    = FSUBZ
C      MAXIMUM DESIGN PRESSURES ALONG THE LENGTH OF THE BAY
C      ARE COMPUTED ON BOTH THE WINDWARD AND LEEWARD SIDES.

```

SWD0232
 SWD0233
 SWD0234
 SWD0235
 SWD0236
 SWD0237
 SWD0238
 SWD0239
 SWD0240
 SWD0241
 SWD0242
 SWD0243
 SWD0244
 SWD0245
 SWD0246
 SWD0247
 SWD0248
 SWD0249
 SWD0250
 SWD0251
 SWD0252
 SWD0253
 SWD0254
 SWD0255
 SWD0256
 SWD0257
 SWD0258
 SWD0259
 SWD0260
 SWD0261
 SWD0262
 SWD0263
 SWD0264
 SWD0265
 SWD0266
 SWD0267
 SWD0268
 SWD0269

```

CALL PRESUR
LPFL      = 2
C
C
C      MINIMUM MECHANICAL PROPERTIES OF HONEYCOMB MATERIALS ARE
      LISTED.
G      = E / (2.0 * (1.0 + AMU))
GFACE  = EFACE / (2. * (1. + AMU))
C      CORE MATERIAL IS 2024-T4 ALUMINUM
CTA    = 40000.
CCA    = 40000.
CSA    = 23000.
ECORE  = E
GCORE  = ECORE / (2. * (1. + AMU))
C      CALCULATE SKIN THICKNESS REQUIRED TO CARRY THE
C      MERIDIANAL COMPRESSIVE LOAD PER INCH
TKMCL  = ANFIMX / FCA
C      CALCULATE SKIN THICKNESS REQUIRED FOR THE
C      CIRCUMFERENTIAL LOAD PER INCH
TKCIL  = PDESMX * DSUBB / (2.0 * FCA * CTH)
C      CALCULATE SKIN THICKNESS REQUIRED FOR SHEAR STRESS
C      DUE TO AERODYNAMIC LIFT
TKSAL  = 2.0 * FZ(1) / (PI * DSUBB * FSA)
C      CALCULATE SKIN THICKNESS REQUIRED FOR HOOP STRESS. IT
C      IS ASSUMED THE PRESSURE DIFFERENTIAL WILL NOT EXCEED 4 PSI
PDIFF  = 4.0
C      CALCULATE EQUIVALENT PANEL RADIUS OF CURVATURE.
ENSUBR = 0.6 * (1.7 - 2.0 * ALB * C7 / DSUBB)
APRC   = 0.5 * DSUBB * ENSUBR / CTH
TKHOOP = FS * PDIFF * APRC / FTA
C      SELECT LARGEST VALUE OF SKIN THICKNESS. USE HALF THIS
C      VALUE AS FACE THICKNESS.
TF     = 0.5 * AMAX1(TKMCL, TKCIL, TKSAL, TKHOOP)
C      NOW COMPARE THE THEORETICAL FACE THICKNESS WITH THE
C      MINIMUM ALLOWABLE FACE THICKNESS.
C      THE FACE THICKNESS MUST BE INCREASED TO MINIMUM ALLOWAS
C      FACE THICKNESS.
IF (TF .LT. TMINC) TF = TMINC
C      NEXT, STANDARD GAUGES ARE TO BE USED IF KTFSTD IS 1.
IF (KTFSTD .NE. 1) GO TO 239

```

```

SWD0309
SWD0310
SWD0311
SWD0312
SWD0313
SWD0314
SWD0315
SWD0316
SWD0317
SWD0318
SWD0319
SWD0320
SWD0321
SWD0322
SWD0323
SWD0324
SWD0325
SWD0326
SWD0327
SWD0328
SWD0329
SWD0330
SWD0331
SWD0332
SWD0333
SWD0334
SWD0335
SWD0336
SWD0337
SWD0338
SWD0339
SWD0340
SWD0341
SWD0342
SWD0343
SWD0344
SWD0345
SWD0346

JF      = 1
237 IF (TSTD(JF) .GT. TF) GO TO 238
IF (JF .GT. 30) GO TO 98
234 JF   = JF + 1
GO TO 237
238 TF   = TSTD(JF)
239 CONTINUE
FFAX(1) = ANFIMX / (2. * TF)
FFC(1)  = (PDESMX * DSUBB) / (4. * TF * CTH)

C
C      CALCULATE CELL SIZE.
CELLWD  = SQRT((2. * TF**2*E)/(C8*AMAX1(FFAX(1),FFC(1))))
NCELL   = CELLWD * 16.
CELSIZ(1) = FLOAT(NCELL) * .0625
IF (CELSIZ(1) .GT. 1.) CELSIZ(1) = 1.
C      THE CORE USED IN THIS ANALYSIS IS ASSUMED TO HAVE A HEXAG
C      ONAL SHAPE. INITIALIZE DIMENSIONAL DATA.
BETADG  = 60.0
BETA    = 0.017453 * BETADG
SBETA   = SIN(BETA)
W       = 0.5 * (A + B)
R       = B/A
B       = A
A       = CELSIZ / (2.0 * SBETA)
CBETA   = COS(BETA)
C      SET FIRST TRIAL RIBBON THICKNESS
TRIBN   = AMAX1(TRBMIN,NF) * .0005
C      IF KTRSTD IS 1, STANDARD RIBBON GAUGES ARE TO BE USED.
IF (KTRSTD .NE. 1) GO TO 224
JG      = 1
222 IF (TRBSTD(JG) .GE. TRIBN) GO TO 223
JG      = JG + 1
IF (JG .GT. 8) GO TO 98
GO TO 222
223 TRIBN = TRBSTD(JG)
224 CONTINUE
XL=ALB
C      ASSUME THE RIBBON DIRECTION : PARALLEL TO THE VEHICLE

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252 CONTINUE
253 CONTINUE
  IF ( H/W .GT. 5.0) GO TO 227
  Y = (9.3 / (H/W)) - (2.1 / (H/W)**2)
  IF ( H/W .LT. 0.5) Y = 11.0
227 CONTINUE
  G1 = GC1 * (1.0 + .01 * Y)
  G2 = GC2 * (1.0 + .01 * Y)
  P = PDESMN
  THIS BAY HAS PASSED A PANEL STABILITY TEST ON THE
  WINDWARD SIDE. NEXT, CHECK LEEWARD SIDE.
  CALL INSTBL
  IF (KG .EQ. 1) GO TO 98
  IF (CRG .GE. ANFIMX) GO TO 254
  IF ( H .GT. 2.0) GO TO 232
  H = H + .125
  GO TO 253
232 CONTINUE
  IF (KTFSTD .EQ. 1) GO TO 234
  TF = 1.1 * TF
  GO TO 239
254 CONTINUE

  THE DESIGN HAS PASSED THE PANEL STABILITY TEST. THE
  PANEL IS NOW CHECKED FOR WRINKLING, USING THE ANALYSIS BY
  BURNS IN LOCKHEED TECHNICAL REPORT 6-62-64-17, DEC. 1964,
  PAGES 3-3 AND 3-4.
241 AFAC = (2.496 * TF / H) * ((EFACE * AMIN1(A,B)) /
1((GORE * TRIBN)**0.33333)
  IF (AFAC .LT. 1.) GO TO 243
  THIN CORE
  AFWRIN = 0.93 * EFACE * ((GORE * TRIBN * TF) /
1(EFACE * AMIN1(A,B) * H)**0.5
  GO TO 244
  THICK CORE
243 AFWRIN = 0.83 * EFACE * ((GORE * TRIBN)/(EFACE * AMIN1(A,B)))
1**0.66667
244 CONTINUE
  COMPARE WRINKLING ALLOWABLE AND APPLIED STRESSES. IF

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C          WRINKLING IS PREDICTED, INCREASE RIBBON THICKNESS.
          IF (AFWRIN - AMAX1(FFAX(1),FFC(1)))246, 247, 247
246 CONTINUE
C          DESIGN IS INADEQUATE. INCREASE RIBBON THICKNESS. RETEST.
          IF (KTRSTD * NE, 1) GO TO 271
          IF (JG * GT, 8) GO TO 98
          JG = JG + 1
          TRIBN = TRBSTD(JG)
          GO TO 241
271 TRIBN = TRIBN + .00001
          GO TO 241
247 CONTINUE
C          PANEL DESIGN FOR THIS BAY IS NOW STRUCTURALLY ADEQUATE.
C
C          COMPUTE BAY DIMENSIONS NEEDED IN WEIGHT CALCULATIONS.
          DSUBB IS OUTSIDE BASE DIAMETER, D(1) IS THE UPPER OUTSIDE
          DIAMETER, DBNSD IS INSIDE BASE DIAMETER AND D(1) IS UPPER
          INSIDE DIAMETER.
          D(1) = DSUBB - (C2)*(ALB)
          D(1) = D(1) - 2. * ((H - TF) / CTH)
          DBNSD = DSUBB - 2. * ((H - TF) / CTH)
C
C          CALCULATE WEIGHT OF INNER FACE.
          WTINFC = (ALB * C5) * (DBNSD + D(1)) * TF *
          1 ( RHO / 2. )
C          CALCULATE WEIGHT OF OUTER FACE
          WTOTFC = (ALB * C5) * (DSUBB + D(1)) * TF *
          1 ( RHO / 2. )
          WFACE(1) = WTINFC + WTOTFC
C
C          CALCULATE CORE WEIGHT
          VCORE = (PI/2. ) * (DBNSD + D(1)) * (ALB / CTH) *
          1 (H - TF)
          ROCORE = (TRIBN * RHO * (1. + R)) / (A * SBETA * (R + CBETA))
          WTCORE(1) = VCORE * ROCORE
C
C          MAKE AN ALLOWANCE FOR THE WEIGHT OF BONDING MATERIAL
          ASSUME ONE OUNCE PER SQ. FT. PER FACE
          WBOND = (PI/2. ) * (DBNSD + D(1)) * (ALB / CTH)

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1 * 0.125 / 144.0
C MAKE AN ALLOWANCE FOR CIRCUMFERENTIAL SPLICE WEIGHT.
  WSPICE(I) = PI * (D(I) + DINS) * (TF * (3.5 + H + TF)) * RHO
C THE REQUIRED MOMENT OF INERTIA OF A STIFFENING RING IS
C CALCULATED.
  CALL IREQ
C STIFFENING RING FOR THE BAY IS DESIGNED USING STANDARD
C SKIN GAUGES AND INPUT NORMALIZED RING CROSS-SECTION.
  K = 0
  256 K = K + 1
  IF (K.GT. 29) GO TO 98
  CALL RING
C CHECK RING TO SEE IF MOMENT OF INERTIA IS ADEQUATE. AND
C IF OUTSTANDING FLANGE WILL BUCKLE.
  CALL RSTRES
  IF (FCFB.LT. FRING) GO TO 256
  IF (AIRING.GT. AIREQ) GO TO 257
  GO TO 256
  257 CONTINUE
  WRING(I) = AST * (DINS - 2. * DD / CTH) * PI * RHO
  WSEG(I) = WFACE(I) + WTCORE(I) + WBOND + WRING(I) + WSPICE(I)
  VSEG(I) = (PI*ALB/12.)*(DSUBB**2 + DSUBB*D(I) + D(I)**2)
  WTDEX(I) = WSEG(I) * 1728. / VSEG(I)
  ALOPT(I) = ALB
  TFACE(I) = TF
  TRING(I) = TSTD(K)
  TCORE(I) = H
  SLOPT = SLOPT + ALB
  SUMAL = SUMAL + ALOPT(I)
  TRBN(I) = TRIBN
  FFS(I) = FZ(I) / (PI * DSUBB * TF)
  FFB(I) = FS * PDIFF * APRC / (2. * TF)
  FFWR(I) = AFWRIN
  FFDMP(I) = (2. * TF**2 * EFACE) / (C8 * CELSZ(I)**2)
C
C BAY WEIGHT IS ADDED TO PREVIOUS BAY WEIGHTS FOR THIS
C FRUSTUM.
  WCONE = WCONE + WSEG(I)

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C      BAY VOLUME WHICH IS USEFUL FOR PAYLOAD IS CALCULATED
C      NEXT. TWO RING HEIGHTS AND TWO PERCENT OF THE BASE DIAMETER
C      ARE SUBTRACTED FROM THE DIAMETERS OF THE CONICAL FRUSTUM IN
C      ORDER TO DETERMINE THIS USEFUL VOLUME.
C      USEFUL BAY VOLUME IS ADDED TO THE SUM OF PREVIOUS BAY
C      VOLUME FOR THIS FRUSTUM.
C      DUSE1 = DSUBB - 2. * AOT * TRING(I) - .02 * DBASE
C      DUSE2 = D(I) - 2. * AOT * TRING(I) - .02 * DBASE
C      VUSE = VUSE + .2617994 * ALOPT(I)*(DUSE1**2 + DUSE1*DUSE2
1      DUSE2**2)/1728.
C
C      BASE DIAMETER OF THE NEXT BAY IS SET EQUAL TO THE TOP
C      DIAMETER OF THE CURRENT BAY.
C
C      DSUBB = D(I)
C
C      THE NUMBER OF BAYS IN THIS FRUSTUM ARE COUNTED.
C
C      NBAY = NBAY + 1
C      A CHECK IS MADE TO DETERMINE IF THIS IS THE LAST BAY
C      OF THE FRUSTUM.
C
C      IF( ALB. GE. ALMX2) GO TO 362
C      THE PROGRAM GOES ON TO DESIGN THE NEXT BAY.
C
C      GO TO 101
362 CONTINUE
C
C      THE LAST BAY OF THE FRUSTUM HAS BEEN DESIGNED. WEIGHT
C      OF THE TOP RING OF THE FRUSTUM IS INCREASED TO ALLOW FOR
C      ATTACHMENT PROVISIONS.
C      WRING(I) = 2. * WRING(I)
C      DATA ON THIS FRUSTUM ARE WRITTEN OUT FOR THE SUMMARY
C      REPORT.
C      WRITE (6,313) NF,DMX,DMIN,THTA(NF),
1ALCONE,ELMIN(NF),TMNC(NF),NBAY,VUSE,WCONC
C      DMX = DMN(NF)
C      GROSS VOLUME OF THE FRUSTUM IS CALCULATED AND ADDED TO
C      THE GROSS VOLUME OF PREVIOUS FRUSTUMS.
C      VGROS = 0.2617994*ALCONE*(DBASE**2+DBASE*DMIN+DMIN**2)
C      VGROSS = VGROSS + VGROS

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C
C      USEFUL VOLUME OF THIS FRUSTUM IS ADDED TO USEFUL VOLUME
C      OF PREVIOUS FRUSTUMS.
C      VTOT = VTOT + VUSE
C      WEIGHT OF THIS FRUSTUM IS ADDED TO WEIGHT OF PREVIOUS
C      FRUSTUMS.
C      WTOT = WTOT + WCONE
C      BASE DIAMETER OF NEXT FRUSTUM IS SET EQUAL TO TOP
C      DIAMETER OF THIS FRUSTUM.
C      DBASE = DMIN
C      TOTAL NUMBER OF BAYS DESIGNED UP TO THIS POINT IS STORED.
C      IMX(NF) = I
C      PROGRAM GOES ON TO NEXT FRUSTUM.
C      129 CONTINUE
C      GO TO 326
C      98 CONTINUE
C
C      CONTROL HAS BEEN TRANSFERRED TO THIS POINT BECAUSE THE
C      INDEX OF ONE OF THE SUBSCRIPTED VARIABLES HAS EXCEEDED THE
C      NUMBER PERMITTED BY THE DIMENSION STATEMENT.
C      WRITE (6,120)
C      WRITE (6,121)
C      WRITE (6,614) I,J,JF,JG,K
C      IF(KG.EQ. 1) WRITE(6,201)GBAR
C      WRITE (6, 501)
C      326 CONTINUE
C      IMAX = 1
C      WRITE(6,314)
C      WRITE (6,315) ALTOT,IMAX,VTOT,WTOT
C      WRITE (6,606)
C      SCAP = 2. * PI * RCAP * ALCAP
C      SURFACE AREA OF THE NOSE CAP IS COMPUTED
C      NOSE CAP SKIN THICKNESS REQUIRED FOR STRUCTURAL INTEGRITY
C      IS DETERMINED AND COMPARED WITH THICKNESS PREVIOUSLY
C      CALCULATED TO MEET THERMAL REQUIREMENTS. THE LARGER OF THE
C      TWO THICKNESSES IS USED TO COMPUTE CAP WEIGHT.
C      CALL TNOSS
C      TCAP = AMAX1(TCAPST, TCAPTH)
C      NOSE CAP WEIGHT IS COMPUTED.

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C      WCAP      = SCAP* RHO * TCAP
      USEFUL VOLUME OF NOSE CAP IS CALCULATED.
C      RUSE      = RCAP - 2. * AOT * TRING(1) - .01 * DBASE
      HUSE
C      VCAP      = ALCAP - 2. * AOT * TRING(1) - .01 * DBASE
      = 1.0471976*HUSE*HUSE*(3.*RUSE - HUSE)/1728.
      GROSS VOLUME OF CAP IS COMPUTED AND ADDED TO GROSS
      VOLUME OF THE FRUSTUMS.
C      VGROSS    = 1.0471976*ALCAP*ALCAP*(3.*RCAP - ALCAP)
      = (VGROSS + VGRS)/ 1728.
      USEFUL VOLUME OF CAP IS ADDED TO USEFUL VOLUME OF CONE
      TO OBTAIN TOTAL USEFUL VOLUME OF FAIRING.
C      VTOT      = VTOT + VCAP
      TOTAL LENGTH OF FAIRING IS OBTAINED BY SUMMING CONE AND
      CAP LENGTHS.
C      TOTAL FAIRING WEIGHT IS THE SUM OF CAP WEIGHT AND CONE
      WEIGHT.
C      WTOT      = WTOT + WCAP
      DATA REPORT IS WRITTEN OUT.
      WRITE (6,82)
      WRITE (6,511) PDSPH
      WRITE (6,67) TMINN
      WRITE (6,96) TMPMAX
      WRITE (6,89) TCAPST
      WRITE (6,90) TCAPTH
      WRITE (6,91) TCAP
      WRITE (6,87) RCAP
      WRITE (6,84) ALCAP
      WRITE (6,88) SCAP
      WRITE (6,316) VCAP
      WRITE (6,94) WCAP
      WRITE (6,92)
      WRITE (6,85) ALTOT
      WRITE (6,86) VTOT
      WRITE (6,329) VGRS
      WRITE (6,95) WTOT
C
C      IF AN INTEGER WAS READ IN FOR KEY, DETAILED DESIGN
      INFORMATION IS WRITTEN OUT FOR THE FAIRING.
C      IF (KEY .LT. 1) GO TO 10

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```

327 CONTINUE
  WRITE (6, 501)
  WRITE (6, 606)
  WRITE (6, 51)
  WRITE (6, 14)
  14 FORMAT (59X, 10HRING DATA ///)
  GO TO ( 1615, 1616, 1617 ), IKIND
1615 WRITE (6, 2611)
2611 FORMAT( 23X, 4HTYPE , 61X, 11HANGLE )
  GO TO 2625
1616 WRITE (6, 2612)
2612 FORMAT( 23X, 4HTYPE , 61X, 11HZEE SECTION )
  GO TO 2625
1617 WRITE (6, 2613)
2613 FORMAT(23X, 4HTYPE , 61X, 11HAT SECTION )
2625 CONTINUE
  WRITE (6, 1620) BOT , AOT, COT, FCFB
1620 FORMAT( 23X, 12HBASE LEG B/T , 53X, 11HBOT
1 23X, 15HUPRIGHT LEG B/T , 50X, 11HAOT
2 23X, 19HOUTSTANDING LEG B/T , 46X, 11HCOT
3 23X, 27HFLANGE BUCKLING STRESS, PSI , 38X, 11HFCFB = F7.0//)
  WRITE (6, 22)
  22 FORMAT (55X, 18HSANDWICH FACE DATA //)
  GO TO (671, 672, 673), MATF
671 WRITE (6, 676)
676 FORMAT (23X, 13HFACE MATERIAL , 36X, 24H2024-T4 ALCLAD ALUMINUM )
  GO TO 679
672 WRITE (6, 677)
677 FORMAT (23X, 13HFACE MATERIAL , 42X, 17H2024-T4 ALUMINUM )
  GO TO 679
673 WRITE (6, 678)
678 FORMAT (23X, 13HFACE MATERIAL , 42X, 17H7075-T6 ALUMINUM )
679 CONTINUE
  WRITE (6, 501)
  WRITE (6, 167)
  WRITE (6, 168)
  WRITE (6, 169)
  WRITE (6, 607)

```

```

NF          = 1
IW          = 1
I1          = 1
I2          = 46
451 CONTINUE
IF (I2.GT. IMAX) I2 = IMAX
DO 453 I = I1, I2
ENFIMN(I) = - ENFIMN(I)
ENFIMX(I) = - ENFIMX(I)
WRITE (6,58) NF, IW, D(I), ALOPT(I), TFACE(I), TRBN(I), TRING(I), SWD0665
1 WFACE(I), WTCORE(I), WRING(I), WTDEX(I), TCORE(I), WSEG(I),
2 CELSZ(I)
IW          = IW + 1
IF (IMX(NF).GT. I) GO TO 453
IW          = 1
NF          = NF + 1
453 CONTINUE
IF (I2.GE. IMAX) GO TO 457
I1          = I2 + 1
I2          = I2 + 54
WRITE (6, 501)
WRITE (6,606)
GO TO 451
457 CONTINUE
WRITE (6, 501)
WRITE (6,806)
WRITE (6,807)
WRITE (6,808)
WRITE (6,809)
NF          = 1
IW          = 1
I1          = 1
I2          = 46
461 CONTINUE
IF (I2.GT. IMAX) I2 = IMAX
DO 463 I = I1, I2
WRITE (6,508) NF, IW, ENFIMN(I), ENFIMX(I), FFAI(I), FFC(I),
1 FFS(I), FFB(I), FFWR(I), FCA, FFDMP(I)
IW          = IW + 1
SWD0655
SWD0656
SWD0657
SWD0658
SWD0659
SWD0660
SWD0661
SWD0662
SWD0663
SWD0664
SWD0665
SWD0666
SWD0667
SWD0668
SWD0669
SWD0670
SWD0671
SWD0672
SWD0673
SWD0674
SWD0675
SWD0676
SWD0677
SWD0678
SWD0679
SWD0680
SWD0681
SWD0682
SWD0683
SWD0684
SWD0685
SWD0686
SWD0687
SWD0688
SWD0689
SWD0690
SWD0691
SWD0692
SWD0693

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```

IF (IMX(NF) .GT. 1) GO TO 463
IW = 1
NF = NF + 1
463 CONTINUE
IF (I2 .GE. IMAX) GO TO 467
I1 = I2 + 1
I2 = I2 + 54
WRITE (6, 501)
WRITE (6, 606)
GO TO 461
467 CONTINUE
C
C      IF 2 WAS READ IN FOR KEY, DETAILED LOADS INFORMATION
C      IS WRITTEN OUT.
IF (KEY .NE. 2) GO TO 10
WRITE (6, 501)
WRITE (6, 606)
WRITE (6, 612)
WRITE (6, 608)
WRITE (6, 609)
WRITE (6, 610)
SUMAL = 0.
IW = 1
NF = 1
I1 = 1
I2 = 46
THETA = 0.0174532925*THTA(NF)
CTH = COS(THETA)
STH = SIN(THETA)
C2 = 2.*STH/CTH
C3 = 0.3/CTH
DSUBB = DBAS
ALB = ALOPT(1)
LPFL = 1
471 IF (IMAX .LT. I2) I2 = IMAX
DO 473 I = I1, I2
CALL PRESUR
LPFL = 2
ANFIAX = 0.5*(ENFIMN(1) + ENFIMX(1))/FS

```

SWD0694
SWD0695
SWD0696
SWD0697
SWD0698
SWD0699
SWD0700
SWD0701
SWD0702
SWD0703
SWD0704
SWD0705
SWD0706
SWD0707
SWD0708
SWD0709
SWD0710
SWD0711
SWD0712
SWD0713
SWD0714
SWD0715
SWD0716
SWD0717
SWD0718
SWD0719
SWD0720
SWD0721
SWD0722
SWD0723
SWD0724
SWD0725
SWD0726
SWD0727
SWD0728
SWD0729
SWD0730
SWD0731

```

ANFIB      = ANFIAX - ENFIMN(I)/FS
AXLOAD     = -ANFIAX*DSUBB*.94248/C3
BEND       = -0.23562*ANFIB*DSUBB**2/C3
WRITE (6,611) NF,IW,DSUBB,D(I),ALOPT(I),SUMAL,PDESMMX,PDESMN,
1 AXLOAD,FZ(I),BEND
1W         = IW + 1
SUMAL      = SUMAL + ALOPT(I)
DSUBB      = D(I)
ALB        = ALOPT(I+1)
IF (IMX(NF) .GT. I) GO TO 473
1W         = 1
NF         = NF + 1
THETA      = 0.0174532925*THTA(NF)
CTH        = COS(THETA)
STH        = SIN(THETA)
C2         = 2.*STH/CTH
C3         = 0.3/CTH
473 CONTINUE
IF (IMAX .LE. I2) GO TO 477
11         = I2 + 1
12         = I2 + 54
WRITE (6,501)
WRITE (6,606)
GO TO 471
477 CONTINUE
WRITE (6,613) DSUBB,SUMAL,AXLDCP,FSBZCP,BNDCAP
GO TO 10
51 FORMAT (48X,36H DESIGN DETAILS OF CONICAL FRUSTUMS ///)
54 FORMAT (54X,24H FAIRING GEOMETRY //)
58 FORMAT(1X,I2,I1H-,I2,4X,F6.1,F8.1,F10.4,F9.5,F9.4,F11.2,
1 F11.2, F9.2, F11.4, F11.4, F11.1, F10.4)
63 FORMAT (48X,33H CONSTRAINTS ON FAIRING DESIGN //)
64 FORMAT (23X,77H MINIMUM BAY LENGTH CONSIDERED,IN,
1 ALMIN = ,F8.3)
66 FORMAT (23X,77H MINIMUM ALLOWABLE THICKNESS OF CONE SKIN,IN,
1 TMINC = ,F8.4)
67 FORMAT (23X,77H MINIMUM ALLOWABLE THICKNESS OF NOSE CAP SKIN,IN,
1 TMINN = ,F8.4)
68 FORMAT (48X,36H SPECIFIED DESIGN PRESSURES //)

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SWD0732
SWD0733
SWD0734
SWD0735
SWD0736
SWD0737
SWD0738
SWD0739
SWD0740
SWD0741
SWD0742
SWD0743
SWD0744
SWD0745
SWD0746
SWD0747
SWD0748
SWD0749
SWD0750
SWD0751
SWD0752
SWD0753
SWD0754
SWD0755
SWD0756
SWD0757
SWD0758
SWD0759
SWD0760
SWD0761
SWD0762
SWD0763
SWD0764
SWD0765
SWD0766
SWD0767
SWD0768
SWD0769
SWD0770

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69 FORMAT (23X,77H DESIGN PRESSURE ON WNDWRD SIDE OF CONE, (SAFETY FASWD0771
1CTOR=1.4), PSI PDSEMX = ,F8.3) SWD0772
71 FORMAT (45X,42H AERODYNAMIC DATA USED IN COMPUTING LOADS //) SWD0773
72 FORMAT (23X,77H DYNAMIC PRESSURE, LBS./SQ. FT. SWD0774
1 QBAR = ,F8.2) SWD0775
73 FORMAT (23X,77H MACH NUMBER AT DESIGN DYNAMIC PRESSURE SWD0776
1 AMACH = ,F8.3) SWD0777
74 FORMAT (23X,77H ANGLE OF ATTACK AT DESIGN DYNAMIC PRESSURE, DEGREESWD0778
1S ALPHA = ,F8.2 ) SWD0779
75 FORMAT (48X, 36H COMPUTED AERODYNAMIC LOADS DATA //) SWD0780
76 FORMAT (23X,77H PRESSURE COEFFICIENT ON CONE AT ZERO ANGLE OF ATTASWD0781
1CK CPO = ,F8.4) SWD0782
82 FORMAT (57X,16H NOSE CAP DESIGN // ) SWD0783
83 FORMAT (23X,77H LENGTH OF CONICAL SECTION, IN. SWD0784
1 ALCON = ,F8.2) SWD0785
84 FORMAT (23X,77H LENGTH OF NOSE CAP, IN. SWD0786
1 ALCAP = ,F8.2) SWD0787
85 FORMAT (23X,77H TOTAL LENGTH OF FAIRING, IN. SWD0788
1 ALTOT = ,F8.2 ) SWD0789
86 FORMAT (23X,77H USEFUL VOLUME OF FAIRING, CU. FEET SWD0790
1 VTOT = ,F8.2 ) SWD0791
87 FORMAT (23X,77H NOSE CAP RADIUS, IN. SWD0792
1 RCAP = ,F8.3 ) SWD0793
88 FORMAT (23X,77H NOSE CAP SURFACE AREA, SQ. IN. SWD0794
1 SCAP = ,F8.2 ) SWD0795
89 FORMAT (23X,77H NOSE CAP SKIN THICKNESS REQUIRED FOR STRUCTURAL INSWD0796
1TEGRITY, IN. TCAPST = ,F8.3 ) SWD0797
90 FORMAT (23X,77H NOSE CAP SKIN THICKNESS REQUIRED FOR THERMAL REASOSWD0798
1NS, IN. TCAPTH = ,F8.3 ) SWD0799
91 FORMAT (23X,77H NOSE CAP SKIN THICKNESS USED TO CALCULATE WEIGHT, SWD0800
1IN. TCAP = ,F8.3 ) SWD0801
92 FORMAT (44X,43H TOTAL LENGTH, VOLUME AND WEIGHT OF FAIRING // ) SWD0802
93 FORMAT (23X,77H WEIGHT OF CONICAL SECTION, LBS. SWD0803
1 WCONE = ,F8.2 ) SWD0804
94 FORMAT (23X,77H WEIGHT OF NOSE CAP, LBS. SWD0805
1 WCAP = ,F8.2 ///) SWD0806
95 FORMAT (23X,77H TOTAL WEIGHT OF FAIRING, LBS. SWD0807
1 WTOT = ,F8.2 ) SWD0808

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96 FORMAT (23X,77H MAXIMUM ALLOWABLE TEMPERATURE OF SKIN, DEG. F. SWD0809
1 TMPMAX = ,F8.1 , SWD0810
120 FORMAT (23X,80H DESIGN OF THE CONICAL SECTION OF THE FAIRING HAS NSWD0811
10T BEEN COMPLETED. THE I, J, , SWD0812
121 FORMAT (23X,80H OR K INDEX HAS EXCEEDED THE MAXIMUM PERMITTED BY TSWD0813
1HE DIMENSION STATEMENT. , SWD0814
131 FORMAT (5F12.8, 216) SWD0815
132 FORMAT (216,4F12.8,112) SWD0816
162 FORMAT (23X,77H AMBIENT PRESSURE AT DESIGN CONDITIONS, PSI SWD0817
1 PSTAT = ,F8.3 ///) SWD0818
163 FORMAT (23X,77H INTERNAL-TO-AMBIENT PRESSURE DIFFERENCE AT DESIGN SWD0819
1CONDITIONS,PSI DELTAP = ,F8.3 , SWD0820
164 FORMAT (23X,77H CONICAL SECTION SKIN THICKNESS REQUIRED FOR THERMASWD0821
1L REASONS, 1N. TCONTH = ,F8.4 ///) SWD0822
165 FORMAT (77X, 9H WINDWARD,27X,8H LEEWARD) SWD0823
167 FORMAT (128H FRUSTUM SHELL BAY FACE RIBBON RING SWD0824
1 TOTAL TOTAL RING WEIGHT SHELL SWD0825
2 CELL , SWD0826
168 FORMAT (128H -BAY O.D. LENGTH GAUGE GAUGE SWD0827
1 FACE WT. CORE WT. WT. INDEX THICKNESS BAY WT. SWD0828
2 WIDTH , SWD0829
169 FORMAT (128H NO. (IN) (IN) (IN) (IN) SWD0830
1 (LB) (LB) (LB/CU FT) (IN) (LB) SWD0831
2 (IN) , SWD0832
170 FORMAT (23X,77H DESIGN PRESSURE ON LEEWRD SIDE OF CONE, (SAFETY FASWD0833
1CTOR=1.4), PSI PDESMN = ,F8.3) SWD0834
178 FORMAT (14X,103H RADIUS OF THE NOSE CAP IS ZERO. FOR THIS CASE THESWD0835
1 HEAT TRANSFER EQUATIONS ARE NOT VALID. THEREFORE, NO/ 9X,70H THERSWD0836
2MAL CONSTRAINTS HAVE BEEN IMPOSED ON THE DESIGN OF THIS FAIRING. SWD0837
3 ///) SWD0838
201 FORMAT (20X,52HCASE TERMINATED SINCE GBAR IS OUTSIDE RANGE. GBAR =SWD0839
1 E11.4 , SWD0840
206 FORMAT (23X,77H CHANGE IN PRESSURE COEFFICIENT DUE TO ANGLE OF ATTSWD0841
1ACK CPA = ,F8.4) SWD0842
307 FORMAT (56X,18H AERODYNAMIC LOADS // ) SWD0843
308 FORMAT (48X,34H CONSTRAINTS ON DESIGN OF FRUSTUMS ///) SWD0844
309 FORMAT (48X,36H DESIGN SUMMARY FOR FRUSTUM SECTION //) SWD0845
310 FORMAT (20X, 92H FRUS-- LARGESWD0846
1E SMALL HALF LENGTH MIN. BAY MIN. NO. USEFUSWD0847

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2L      WEIGHT )
311  FORMAT (20X,
1.      DIA.      ANGLE      92H  TUM      SWD0848
2E      )
312  FORMAT (20X,
1)      (IN)      (DEG)      (IN)      GAUGE  OF  DIASWD0849
2)      (LB)      )
313  FORMAT(21X,14,
1  F10.4,I7,F11.2,F11.2)
314  FORMAT (57X, 6H -----,24X,25H  VOLUMSWD0850
315  FORMAT (46X,6HTOTALS,F11.1,23X,14,2F11.2 )
316  FORMAT (23X,77H USEFUL VOLUME OF NOSE CAP, CU. FT.
1      VCAP = ,F8.2 )
329  FORMAT (23X,77H TOTAL VOLUME OF FAIRING, CU. FEET
1      VGROSS = ,F8.2 )
365  FORMAT (3F12.8,35X,11)
501  FORMAT (1H1)
508  FORMAT (1X,I2,I1H-,I2,4X,F8.2,4X,F8.2,5X,F7.0,F9.0,2F10.0,5X,F9.0,
1  4X,F7.0,5X,F7.0)
511  FORMAT (23X,77H DESIGN PRESSURE ON NOSE CAP, PSI
1      PDSPH = ,F8.3 )
550  FORMAT (42X,48HDATA REPORT FOR OPTIMIZED NOSE FAIRING STRUCTURE /)SWD0868
555  FORMAT (59X,12HCASE NUMBER ,I3 //)
606  FORMAT (1X, //)
607  FORMAT (1X, //)
608  FORMAT (10X,112H FRUSTUM  BAY BASE  BAY TOP  BAY  DISTANCE SWD0873
1  DES. PRES. DES. PRES. AXIAL  SHEAR  BENDING )SWD0874
609  FORMAT (10X,112H -BAY  DIA.  DIA.  LENGTH  FROM BASESWD0875
1  WNDWRD  LEWRD  LOAD  LOAD  MOMENT ;SWD0876
610  FORMAT (10X,112H NO.  (IN)  (IN)  (IN)  (IN) SWD0877
1  (PSI)  (LBS)  (LBS)  (IN-LBS) //)SWD0878
611  FORMAT (10X,I2,I1H-,I2,F12.1,F10.1,F9.1,F11.1,F12.2,F13.2,F13.1,
1  F12.1,F14.1)
612  FORMAT (54X,24H  DETAILED LOADS DATA  ///)
613  FORMAT (17H
1  F14.1)
614  FORMAT (22X,93H DESIGN OF THE FAIRING HAS NOT BEEN COMPLETED. ONE SWD0880
1  OF THE SUBSCRIPTED VARIABLES HAS EXCEEDED /35X,20H THE MAXIMUM VALSWD0881
SWD0882
SWD0883
SWD0884
SWD0885

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2UE. /11X,13H I(MAX) = 400 ,6X,12H J(MAX) = 29 ,7X,13H JF(MAX) = 29SWD0886
3 ,6X,13H JG(MAX) = 8 ,7X,13H K(MAX) = 29 ,7X,12H M(MAX) = 7 / SWD0887
4 16X,3H1 = ,14, 11X,3HJ = ,13,12X,4HJF = ,13,11X,4HJG = ,12, 12X, SWD0888
5 3HK = ,13,12X,3HM = ,12 / ) SWD0889
650 FORMAT (1H1) SWD0890
806 FORMAT( 39X, 28HMAX APPLIED FACE STRESS, PSI , 15X, SWD0891
1 21HALLOWABLE STRESS, PSI // ) SWD0892
807 FORMAT (1X,7HFRUSTUM, 3X, 8HWARD, 4X, 7HLEEW, 49X, 4HFACE, SWD0893
1 8X, 4HFACE , 7X, 4HFACE ) SWD0894
808 FORMAT ( 5H -BAY , 5X, 99HLINE LOAD LINE LOAD AXIAL CIRC SWD0895
1 SHEAR BURST WRINKLING YIELD DIMPLING ) SWD0896
809 FORMAT (4H NO. , 7X, 20H(LB/IN) (LB/IN) // ) SWD0897
989 FORMAT(15,6E12,8) SWD0898
990 FORMAT (16, 5F12,4, 16) SWD0899
END SWD0900
$IBFTC C53S2 DECK PTY 001
SUBROUTINE PROPTY (MAT, E, AMU, RHO, TMPMAX) PTY 002
GO TO (1,2,3,4,5 ) , MAT PTY 003
1 CONTINUE PTY 004
C ALUMINUM PROPERTIES (MAT = 1) PTY 005
WRITE (6,141) PTY 006
141 FORMAT (54X,24H MATERIAL = ALUMINUM ///) PTY 007
E = 10500000.0 PTY 008
AMU = 0.3 PTY 009
RHO = 0.1 PTY 010
TMPMAX = 600.0 PTY 011
RETURN PTY 012
2 CONTINUE PTY 013
C MAGNESIUM PROPERTIES (MAT = 2) PTY 014
WRITE (6,142) PTY 015
142 FORMAT (54X,24H MATERIAL = MAGNESIUM ///) PTY 016
E = 6500000.0 PTY 017
AMU = 0.34 PTY 018
RHO = 0.065 PTY 019
TMPMAX = 700.0 PTY 020
RETURN PTY 021
3 CONTINUE PTY 022
C TITANIUM PROPERTIES (MAT = 3) PTY 023

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WRITE (6,143)
143 FORMAT (54X,24H MATERIAL = TITANIUM ///)
E = 16000000.0
AMU = 0.3
RHO = 0.16
TMPMAX = 900.0
RETURN
4 CONTINUE
C STAINLESS STEEL PROPERTIES (MAT = 4)
WRITE (6,144)
144 FORMAT (54X,24H MATERIAL = STEEL ///)
E = 30000000.0
AMU = 0.3
RHO = 0.283
TMPMAX = 1100.0
RETURN
5 CONTINUE
C LOCKALLOY PROPERTIES (MAT = 5)
WRITE (6,145)
145 FORMAT (54X,24H MATERIAL = LOCKALLOY ///)
E = 27000000.0
AMU = 0.3
RHO = 0.076
TMPMAX = 700.0
RETURN
END
$IBFTC C53S3 DECK
SUBROUTINE THERML
C THIS SUBROUTINE COMPUTES SKIN THICKNESSES REQUIRED TO
C KEEP NOSE CAP AND TOP FRUSTUM SKIN TEMPERATURES UNDER TMPMAX.
DIMENSION AT(50),AS(20)
COMMON /TNOS/TMINN,PDSPPH,TCAPST,RCAP
COMMON /THRML/ TMPMAX,MAT,TCONTH,TCAPTH,THETA
C THE EQUATIONS AND STORED COEFFICIENTS IN THIS SUBROUTINE
C WERE OBTAINED BY MEANS OF A MULTIPLE REGRESSION ANALYSIS OF
C DATA RESULTING FROM A THERMAL ANALYSIS OF SPHERES FLYING A
C NOMINAL LLSV TRAJECTORY. THE HOTTEST POINT ON THE NOSE CAP
C IS AT THE STAGNATION POINT OF THE SPHERE, AND THE HOTTEST
C POINT ON THE TOP FRUSTUM IS AT THE TANGENCY POINT OF THE

```

PTY 024
 PTY 025
 PTY 026
 PTY 027
 PTY 028
 PTY 029
 PTY 030
 PTY 031
 PTY 032
 PTY 033
 PTY 034
 PTY 035
 PTY 036
 PTY 037
 PTY 038
 PTY 039
 PTY 040
 PTY 041
 PTY 042
 PTY 043
 PTY 044
 PTY 045
 PTY 046
 PTY 047
 PTY 048
 PTY 049
 TML 001
 TML 002
 TML 003
 TML 004
 TML 005
 TML 006
 TML 007
 TML 008
 TML 009
 TML 010
 TML 011
 TML 012
 TML 013

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C      SPHERICAL NOSE CAP AND TOP FRUSTUM. THE METHOD USED TO OBTAIN TML 014
C      THE ORIGINAL THERMAL DATA IS DESCRIBED IN LMSC DOCUMENT NO. TML 015
C      TM 54-20-7. TML 016
C      TANGENCY POINT COEFFICIENTS FOR ALUMINUM. TML 017
C      DATA AT/      .027903, -.010162, .691498, -.013716, 2.108521, TML 018
C      1      .25293, .52256, -.560318, .896866, -.512848, TML 019
C      TANGENCY POINT COEFFICIENTS FOR MAGNESIUM. TML 020
C      2      -.176254, -1.175537, 1.972534, -.051082, 2.988892, TML 021
C      3      -.392456, 1.529938, -1.154861, .88208, -.055298, TML 022
C      TANGENCY POINT COEFFICIENTS FOR TITANIUM. TML 023
C      4      .056461, .062469, .579819, -.018108, 2.341248, TML 024
C      5      .206238, .399368, -.336502, .835144, -.355804, TML 025
C      TANGENCY POINT COEFFICIENTS FOR STAINLESS STEEL. TML 026
C      6      .194916, 1.457764, -1.19165, -.064727, 1.619629, TML 027
C      7      -.032959, -.851807, .946594, 1.026367, -.943359, TML 028
C      TANGENCY POINT COEFFICIENTS FOR LOCKALLOY. TML 029
C      8      .352531, 1.183907, .312973, .001026, 1.551514, TML 030
C      9      .084351, -.735126, -.335369, .614578, -.366287, TML 031
C      STAGNATION POINT COEFFICIENTS FOR ALUMINUM. TML 032
C      DATA AS/-.014267,5.291153,-1.559019,-.010812, TML 033
C      STAGNATION POINT COEFFICIENTS FOR MAGNESIUM. TML 034
C      1      -.029125,6.776562,-1.966431,-.033269, TML 035
C      STAGNATION POINT COEFFICIENTS FOR TITANIUM. TML 036
C      2      -.001164,5.252344,-0.926601,-.012582, TML 037
C      STAGNATION POINT COEFFICIENTS FOR STAINLESS STEEL. TML 038
C      3      0.009530,3.505341,-1.058136,-.017291, TML 039
C      STAGNATION POINT COEFFICIENTS FOR LOCKALLOY. TML 040
C      4      -.008722,3.676389,-1.086252,-.000541, TML 041
C      = (SIN(THETA))**2 TML 042
C      = 1.0/SQRT(RCAP) TML 043
C      = 0.001*TMPMAX*X2 TML 044
C      = (0.001 * TMPMAX+ 0.46) ** 4 TML 045
C      = X2*S2 TML 046
C      = X5*(0.001*TMPMAX) TML 047
C      = 1.0/(RCAP**.2) TML 048
C      = .001*TMPMAX*X7 TML 049
C      = X7*S2 TML 050
C      = 0.001*TMPMAX*X9 TML 051

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      L      = 1 + 4*(MAT - 1)
      M      = 1 + 10*(MAT - 1)
      YTAN   = AT(M)+X2*AT(M+1)+X3*AT(M+2)+X4*AT(M+3)+X5*AT(M+4)
1      YSTG  = AT(M)+X2*AT(M+1)+X3*AT(M+2)+X4*AT(M+3)+X5*AT(M+4)
      YSTG   = AS(L)+X2*AS(L+1)+X3*AS(L+2)+X4*AS(L+3)
      TCAPTH = 100.*YSTG/(TMPMAX-70.)
      TCONTH = 100.*YTAN/(TMPMAX-70.)
      RETURN
      END
$1BFTC C53S4  DECK
SUBROUTINE AERO
C      THIS SUBROUTINE COMPUTES PRESSURE COEFFICIENTS AT ZERO
C      ANGLE-OF-ATTACK AND THE CHANGE IN PRESSURE COEFFICIENT DUE
C      TO ANGLE-OF-ATTACK. FOR THE PURPOSE OF COMPUTING THESE
C      COEFFICIENTS, EACH FRUSTUM IS TREATED AS A COMPLETE CONE.
C      FIRST, THE PRESSURE COEFFICIENT AT ZERO ANGLE OF ATTACK
C      IS CALCULATED BY THE METHOD OF SIMON AND WALTER, AIAA JOURNAL,
C      JULY 1963, PP 1696-97. THE METHOD APPROXIMATES EXACT
C      SOLUTIONS WITH A MODIFIED QUADRATIC IN SINE SQUARE OF DELTA
C      WITH COEFFICIENTS BEING FUNCTIONS OF GAMMA AND MACH NUMBER.
      COMMON/AERPRS/CPAA,CPOO,LPFL
      COMMON/CONFG/THTA(10),ALF(10),NFMX,DBAS,DMN(10),ICONFG
      COMMON/AERLOD/ALPHA,AMACH,NF
      IF (THTA(NF)) 30,30,31
30  CPOO   = 0.
      CPAA  = 0.
      GO TO 300
31  CONTINUE
      DELTA = 0.0174532925*THTA(NF)
      G=GAMMA=1.4, GAM1=(G+7)/(G+1), GAM2=(G+7)/4, GAM3=((G-1)/4)**2
      GAM4=G/2, GAM5=GAM4*GAM1, GAM6=GAM1/2
      GAM1=3.5
      GAM2=2.1
      GAM3=.01
      GAM4=.7
      GAM5=2.45
      GAM6=1.75
      EM    = AMACH
      IF(EM-1.05)100,1,1

```

TML 052
 TML 053
 TML 054
 TML 055
 TML 056
 TML 057
 TML 058
 TML 059
 TML 060
 AER 001
 AER 002
 AER 003
 AER 004
 AER 005
 AER 006
 AER 007
 AER 008
 AER 009
 AER 010
 AER 011
 AER 012
 AER 013
 AER 014
 AER 015
 AER 016
 AER 017
 AER 018
 AER 019
 AER 020
 AER 021
 AER 022
 AER 023
 AER 024
 AER 025
 AER 026
 AER 027
 AER 028
 AER 029
 AER 030

```

1 FINK=DELTA*57.2958
  IF(FINK-50.)3,3,4
4 WRITE (6,5) FINK
5 FORMAT(84HOSUBR RAP-AT ANGLES BEYOND 50 DEG THE CP OBTAINED IS LIKAER
  IELY TO BE ERRONEOUS. DELTA=F6.2.6H, EMO=F6.2.7H, X=F8.2)
  EM=0.
  RETURN
3 SDEL= SIN (DELTA)
  SSDEL=SDEL*SDEL
  GSSDE=1.4*SSDEL
  EMSTR=SQRT ((1.+GSSDE)/(1.-GSSDE))
22 IF(EM-EMSTR)110,2,2
2 EMS=EM*EM
  EM4=EMS*EMS
  EM6=EM4*EMS
  F1=((EMS-1.)/(EM4*SDEL))+ (6./EM6)+GAM2-GAM3
  Q=1.+(1./EM6)
  Q1=1./EMS
  F1SSD=F1*SSDEL
  F2=GAM6*(1.-Q1)*Q
  F3=GAM5*(1.+Q1)*Q
  SQUID=(F2-F1SSD)**2-((F3-F1)*SSDEL)**2
  CPOO = 0.5*((F2+F1SSD)-SQRT(SQUID))

      NEXT, THE INCREASE IN PRESSURE COEFFICIENT (ON THE HIGH
      PRESSURE SIDE OF THE CONE) DUE TO ANGLE OF ATTACK IS
      COMPUTED. THIS CALCULATION IS BASED ON DATA FROM CHART 8
      OF NACA REPORT 1135 AT MACH 1.5, AND A CIRCUMFERENTIAL
      PRESSURE DISTRIBUTION WHICH IS SINUSOIDAL. THE EQUATION
      FOR CPAA IS VALID FOR MACH NUMBERS BETWEEN 1.4 AND 1.6.
      CHECK IF MACH NUMBER LIES BETWEEN 1.4 AND 1.6.

      IF (AMACH - 1.4) 201,202,203
203 IF (AMACH - 1.6) 202,202,201
201 WRITE (6,205)
      WRITE (6,204)
      WRITE (6,205)
205 FORMAT (1H1)

```

AER 031
 AER 032
 AER 033
 AER 034
 AER 035
 AER 036
 AER 037
 AER 038
 AER 039
 AER 040
 AER 041
 AER 042
 AER 043
 AER 044
 AER 045
 AER 046
 AER 047
 AER 048
 AER 049
 AER 050
 AER 051
 AER 052
 AER 053
 AER 054
 AER 055
 AER 056
 AER 057
 AER 058
 AER 059
 AER 060
 AER 061
 AER 062
 AER 063
 AER 064
 AER 065
 AER 066
 AER 067
 AER 068

```

204 FORMAT (16X,100HMACH NUMBER LIES OUTSIDE THE INTERVAL FROM 1.4 TO 1.6, VALUES CALCULATED FOR CPA MAY BE INACCURATE. )
11.6.
C
C
C
      CALCULATE INCREASE IN THE PRESSURE COEFFICIENT.
202 CPAA      = (2.03 - 1.20*DELTA)*ALPHA*2.*SIN(DELTA)/COS(DELTA)
300 CONTINUE
      RETURN
100 WRITE (6,101) EM
101 FORMAT(32H0      SUBROUTINE RAP--MACH NO.(F6.2,26H) IS LESS THAN 1AER 078
      1.05 EM0=F6.2,7H, EM1=F6.2,5H, X=F8.2)
      EM=0.
      RETURN
110 WRITE (6,111) FINK, EM
111 FORMAT(27H0      SUBROUTINE RAP--DELTA(F6.2,42H) EXCEEDS MAX ANGLE FAER 083
      1OR THE MACH NO. USED(F6.2,8H). EM0=F6.2,7H, EM1=F6.2,4H, X=F8.2)AER 084
      EM=0.
      RETURN
      END
$IBFTC C53S5 DECK
      SUBROUTINE DIAM (XN,DLOC)
      INPUT PARAMETERS--XN,DMN(NF) FOR NF = 1 TO NFMAX,THTA(NF)
      FOR NF = 1 TO NFMAX,NFMAX,ALTOT
      OUTPUT VARIABLES--DLOC
      COMMON/CONF/THTA(10),ALF(10),NFMAX,DBAS,DMN(10),ICONFG
      COMMON/PRESR/CPA(121),CPO(121),XOD(121),DELTAP,FS,LTMAX,SUMAL,
      1 QBAR,ALTOT
      XTOT = ALTOT
      DO 10 N1 = 1,NFMAX
      XTOT = XTOT - ALF(N1)
      DELTA = XN- XTOT
      IF (DELTA) 10,20,20
      10 CONTINUE
      20 CONTINUE
      ANGLE = 0.0174532925*THTA(N1)
      DLOC = DMN(N1) + 2.*DELTA*SIN(ANGLE)/COS(ANGLE)
      RETURN
      END
$IBFTC C53S6 DECK

```

```

SUBROUTINE LOAD
  THIS SUBROUTINE COMPUTES AXIAL LOADS, SHEAR LOADS,
  BENDING MOMENTS AND THE RESULTING LINE LOADS USING EITHER
  THE INPUT PRESSURE DATA OR THAT COMPUTED IN SUBROUTINE AERO.

  COMMON /DLOAD/D1,D2,XOD1,XOD2,CPO1,CPO2,CPA1,CPA2,A3,A4,QB,DP,
1  FSBZ,BND,AXLOD
  COMMON ANFIMN,ANFIMX,DSUBB,PDESMN,PDESMX,E
  COMMON/LOADS/CDCAP,CNCAP,XBCAP,FSUBZ,AXLDCP,FSBZCP,BNDCAP,LTRIG,
1  ALCAP
  COMMON/PRESR/CPA(121),CPO(121),XOD(121),DELTAP,FS,LTMAX,SUMAL,
1  QBAR,ALTOT
  COMMON/CONFG/THTA(10),ALF(10),NFMAX,DBAS,DMN(10),ICONFG
  COMMON/AERLOD/ALPHA,AMACH,NF
  COMMON /CHKLD/ C1,C3,C4,ALB,DELTAS,CHK,RAXMIN,RAXMAX,RPMIN,RPMAX,
1  CHKWND,CHKLEE,C2
  COMMON/PRSLOD/ LTJNCT(10)

  DETERMINE IF THIS IS THE FIRST BAY.

  IF (LTRIG) 399,399,399
399 CONTINUE
  NX = NFMAX
  LTJNCT(NX)= 1
  XOD(1) = ALCAP/DBAS
  XODSTP = (ALCAP + ALF(NFMAX))/DBAS
  DO 458 LTX= 2,LTMAX
  IF ((XOD(LTX) - XOD(LTX-1)).GT..000001) GO TO 458
  NX = NX - 1
  LTJNCT(NX)= LTX
  XOD(LTX) = XODSTP
  XOD(LTX-1)= XODSTP
  XODSTP = XODSTP + ALF(NX)/DBAS
458 CONTINUE
  XOD(LTMAX)= ALTOT/DBAS
  QB = QBAR/144.
  A3 = 1.5707963*QB*DBAS
  A4 = A3*DBAS

```

```

C
C
C
C
C
C
C
DP      = DELTAP
        SINCE THIS IS THE FIRST BAY, THE BENDING MOMENT, AXIAL
        LOAD AND SHEAR LOAD AT THE BASE OF THE FAIRING ARE COMPUTED
        USING THE PRESSURE DATA FOR THE ENTIRE FAIRING.
        THE LOADS CONTRIBUTED BY THE CONICAL FRUSTUMS ARE
        COMPUTED FIRST.
LTRIG   = 1
X       = 0.
AXLOAD  = 0.
BEND    = 0.
FSUBZ   = 0.
EL      = 0.
N1      = 1
LTMX1   = LTMX + 1
LTR1    = 1
DO 463 N1 = 1,NFMAX
LTR2     = LTMX - LTJUNCT(N1)
THETA    = 0.0174532925*THTA(N1)
C2       = 2.*SIN(THETA)/COS(THETA)
DO 371 LTR= LTR1,LTR2
LT       = LTMX1 - LTR
XN       = XOD(LTR)*DBAS
CALL DIAM (XN,D1)
XN       = XOD(LTR-1)*DBAS
CALL DIAM (XN,D2)
XOD1     = XOD(LTR)
XOD2     = XOD(LTR-1)
CPO1     = CPO(LTR)
CPO2     = CPO(LTR-1)
CPA1     = CPA(LTR)
CPA2     = CPA(LTR-1)
CALL DLOD
EL       = EL+X
X        = (XOD1-XOD2)*DBAS
AXLOAD   = AXLOD + AXLOAD
FSUBZ    = FSUBZ + FSBZ
BEND     = BEND + EL*FSBZ + BND

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```

371 CONTINUE
   LTR1 = LTR2 + 1
463 CONTINUE
C
C      THE LOADS CONTRIBUTED BY THE SPHERICAL NOSE CAP ARE
C      ADDED TO THE FRUSTUM LOADS.
C
      AC = .7853982*DMN(NFMAX)**2
      EL = EL + X
      IF (CDCAP) 435,435,436
435 CONTINUE
      X = DMN(NFMAX)/C2
      XBCAP = X/3.
      CZALFA = 2.03 - 1.2*THETA
      CNCAP = CZALFA
      CPSTG = (.166.92158*AMACH**7./((7.*AMACH**2-1.)*2.5-1.))
      STHTA = SIN(THETA)
      CDCAP = CPSTG*(1.+STHTA**2)/2.
436 CONTINUE
      XBAR = XBCAP
      AXLOD = AC*(CDCAP*QB - DELTAP)
      FSBZ = AC*CNCAP*ALPHA*QB
      AXLDCP = AXLOD
      FSBZCP = FSBZ
      BNDCAP = FSBZ*XBAR
      AXLOAD = AXLOD + AXLOAD
      BEND = BEND + (EL + XBAR)*FSBZ
      FSUBZ = FSBZ + FSBZ
C
C      PARAMETERS ARE INITIALIZED FOR USE IN COMPUTING UPPER
C      BAY LOADS.
C
      LFLG = 1
      THETA = 0.0174532925*THTA(NF)
      C3 = 0.3 / COS(THETA)
      C2 = 2.*SIN(THETA)/COS(THETA)
      C21 = C2

```

LDS 079
 LDS 080
 LDS 081
 LDS 082
 LDS 083
 LDS 084
 LDS 085
 LDS 086
 LDS 087
 LDS 088
 LDS 089
 LDS 090
 LDS 091
 LDS 092
 LDS 093
 LDS 094
 LDS 095
 LDS 096
 LDS 097
 LDS 098
 LDS 099
 LDS 100
 LDS 101
 LDS 102
 LDS 103
 LDS 104
 LDS 105
 LDS 106
 LDS 107
 LDS 108
 LDS 109
 LDS 110
 LDS 111
 LDS 112
 LDS 113
 LDS 114
 LDS 115
 LDS 116

```

CP02      = CP0(LTMAX)
CPA2      = CPA(LTMAX)
XOD2      = XOD(LTMAX)
D2        = DBAS
LT        = LTMAX - 1
GO TO 386
393 CONTINUE
XOD8      = (ALTOT - SUMAL)/DBAS
GO TO (445,446,447),LFLG
446 CONTINUE
LFLG      = 1
BNDLB     = BEND
XODLB     = XOD2
CPOLB     = CP02
CPALB     = CPA2
D2LB      = D2
LTLB      = LT
FZLB      = FSUBZ
AXLLB     = AXLOAD
GO TO 445
447 CONTINUE
BEND      = BNDLB
FSUBZ     = FZLB
AXLOAD    = AXLLB
LFLG      = 1
XOD2      = XODLB
CP02      = CPOLB
CPA2      = CPALB
D2        = D2LB
LT        = LTLB
445 CONTINUE

      SINCE THIS IS AN UPPER BAY, THE LOADS AT THE BASE OF
      THIS BAY ARE DETERMINED BY FIRST COMPUTING THE LOADS PRODUCED
      BY THE PRESSURE PROFILE BETWEEN THIS BAY AND THE PREVIOUS
      BAY, AND THEN SUBTRACTING THESE COMPUTED LOADS FROM THE
      LOADS OF THE PREVIOUS BAY.

XOD1      = XOD2

```

```

CP01 = CP02
CPA1 = CPA2
D1 = D2
IF (XOD(LT) - XODB) 387,388,388
387 CONTINUE
XOD2 = XODB
CP02 = CP0(LT) + (CP0(LT+1) - CP0(LT))*(XODB - XOD(LT))
1 / (XOD(LT+1) - XOD(LT))
CPA2 = CPA(LT) + (CPA(LT+1) - CPA(LT))*(XODB - XOD(LT))
1 / (XOD(LT+1) - XOD(LT))
GO TO 389
388 CONTINUE
XOD2 = XOD(LT)
CP02 = CP0(LT)
CPA2 = CPA(LT)
LT = LT - 1
389 CONTINUE
XN1 = XOD1*DBAS
XN2 = XOD2*DBAS
X = XN1-XN2
CALL DIAM (XN1,D1)
CALL DIAM (XN2,D2)
CALL DLOD
FSUBZ = FSUBZ - FSBZ
AXLOAD = AXLOAD - AXLOAD
BEND = BEND - BND - X*FSUBZ
IF (XOD2 - XODB) 386,386,445
386 CONTINUE

      USING THE LOADS COMPUTED ABOVE, LINE LOADS WITH THE
      FACTOR OF SAFETY APPLIED ARE COMPUTED FOR BOTH THE WINDWARD
      AND LEEWARD SIDES OF THE BAY.

ANFIAX = C3*AXLOAD/(0.94248*DSUBB)
ANFIB = BEND*C3 / (0.23562* DSUBB **2)
ANFIMN = FS*(ANFIAX - ANFIB)
ANFIMX = FS*(ANFIAX + ANFIB)
RETURN

```

```

END
$IBFTC C53S7  DECK
SUBROUTINE DLOD
C
C   INPUT VARIABLES--D1,D2,XOD1,XOD2,CP01,CP02,CPA1,CPA2,A3,A4,
C   QB,DP
C
C   OUTPUT VARIABLES-- FSBZ,BND,AXLOD
COMMON /DLOAD/D1,D2,XOD1,XOD2,CP01,CP02,CPA1,CPA2,A3,A4,QB,DP,
1  FSBZ,BND,AXLOD
DX1 = XOD1 - XOD2
IF (DX1.LE..002)GO TO 10
DX2 = XOD1**2 -XOD2**2
DX3 = XOD1**3 - XOD2**3
DX4 = XOD1**4 -XOD2**4
B2 = (CP01 - CP02)/DX1
B1 = CP01 - B2*XOD1
B4 = (CP01 - CP02)/(D1 - D2)
B3 = CP01- B4*D1
A2 = (CPA1 -CPA2)/DX1
A1 = CPA1 - A2*XOD1
AD2 = (D1-D2)/DX1
AD1 = D1 - AD2*XOD1
BTA1 = A1*AD1
BTA2 = 0.5*(A1*AD2 + A2*AD1)
BTA3 = A2*AD2/3.
FSBZ = A3*(BTA1*DX1 + BTA2*DX2 + BTA3*DX3)
BND = A4*(BTA1*DX2/2. - BTA1*XOD2*DX1 + BTA2*DX3/3.
1  -BTA2*DX1*XOD2**2 + BTA3*DX4/4. - BTA3*DX1*XOD2**3)
DD2 = (D1**2 - D2**2)/2.
DD3 = (D1**3 - D2**3)/3.
AXLOD = 1.5707963*(DD2*(QB*B3-DP) + DD3*B4*QB)
GO TO 20
10 DAV = 0.5*(D1 + D2)
CPOAV = 0.5*(CP01 + CP02)
FSBZ = A3*CPOAV*DX1*DAV
CPAAV = 0.5*(CPA1 + CPA2)
BND = A3*DX1*DX1*DAV*(CPA1/2.+2.*(CPA2-CPA1)/3.)
AXLOD = 1.5707963*(D1-D2)*DAV*(CPOAV* QB-DP)
20 CONTINUE
RETURN

```

LDS 194
 DLD 001
 DLD 002
 DLD 003
 DLD 004
 DLD 005
 DLD 006
 DLD 007
 DLD 008
 DLD 009
 DLD 010
 DLD 011
 DLD 012
 DLD 013
 DLD 014
 DLD 015
 DLD 016
 DLD 017
 DLD 018
 DLD 019
 DLD 020
 DLD 021
 DLD 022
 DLD 023
 DLD 024
 DLD 025
 DLD 026
 DLD 027
 DLD 028
 DLD 029
 DLD 030
 DLD 031
 DLD 032
 DLD 034
 DLD 033
 DLD 035
 DLD 036
 DLD 037
 DLD 038

```

END
$1BFTC C53S8 DECK
SUBROUTINE PRESUR
C
C      THIS SUBROUTINE COMPUTES THE MAXIMUM DESIGN PRESSURE
C      OCCURRING ANYWHERE ALONG THE LENGTH OF THE BAY ON BOTH THE
C      WINDWARD AND LEEWARD SIDES OF THE BAY, USING EITHER THE
C      INPUT PRESSURE DATA OR THAT COMPUTED IN SUBROUTINE AERO.
COMMON/AERLOD/ALPHA, AMACH, NF
COMMON /PRSL0D/ LTJNCT(10)
COMMON ANFIMN,ANFIMX,DSUBB,PDESMN,PDESMX,E
COMMON/AERPRS/CPAA,CP00,LPFL
COMMON/PRESR/CPA(121),CPO(121),XOD(121),DELTAP, FS,LTMAX,SUMAL,
1 QBAR,ALTOT
COMMON/CONF/THTA(10),ALF(10),NFMAX,DBAS,DMN(10),ICONFG
COMMON /CHKLD/ C1,C3,C4,ALB,DELTAS,CHK,RAXMIN,RAXMAX,RPMIN,RPMAX,
1 CHKWND,CHKLEE,C2
GO TO (399,400,417,415),LPFL
399 CONTINUE
XODB = ALTOT/DBAS
XOD1 = XODB
QB = QBAR/144.
LT1 = LTMAX + 1
CPMX1 = CPO(LTMAX) + CPA(LTMAX)
CPMN1 = CPO(LTMAX) - CPA(LTMAX)
GO TO 415
400 CONTINUE
L2 = LTMAX
IF (NF.GT.1) L2 = LTJNCT(NF-1) - 1
L1 = LTJNCT(NF)
XODB = (ALTOT - SUMAL)/DBAS
XOD1 = XODB
IF (XOD1.GE.XOD(L2)) XOD1 = XOD(L2) - .00001
DO 412 LT1 = L1,L2
IF (XOD1.LT.XOD(LT1)) GO TO 414
412 CONTINUE
414 LT = LT1
CP00 = CPO(LT-1) + (CPO(LT) - CPO(LT-1))*(XOD1 - XOD(LT-1))
1 / (XOD(LT) - XOD(LT-1))

```

```

CPAA      = CPA(LT-1) + (CPA(LT) - CPA(LT-1))*(XOD1 -XOD(LT-1))
1          /((XOD(LT) - XOD(LT-1))
CPMX1     = CPOO + CPAA
CPMN1     = CPOO - CPAA
415 CONTINUE
LT        = LT1
CPMX1     = CPMX1
CPMN1     = CPMN1
ALN       = 0.
ALX       = 0.
417 CONTINUE
XOD2      = (ALTOT - SUMAL - ALB)/DBAS
IF (XOD2.LE.XOD(L1)) XOD2 = XOD(L1) + .00001
433 LT    = LT - 1
IF (LT.LE.L1) GO TO 419
IF (XOD(LT) - XOD2) 419,422,422
419 CONTINUE
LT        = LT + 1
CPAA      = CPA(LT-1) + (CPA(LT) - CPA(LT-1))*(XOD2 -XOD(LT-1))
1          /((XOD(LT) - XOD(LT-1))
CPOO     = CPO(LT-1) + (CPO(LT) - CPO(LT-1))*(XOD2 - XOD(LT-1))
1          /((XOD(LT) - XOD(LT-1))
CPX      = CPOO + CPAA
CPN      = CPOO - CPAA
JFIN     = 0
GO TO 428
422 CONTINUE
CPN      = CPO(LT) - CPA(LT)
CPX      = CPO(LT) + CPA(LT)
JFIN     = 1
428 IF (CPMX - CPX) 424,425,425
424 CONTINUE
CPMX     = CPX
ALX      = (XODB - XOD(LT))*DBAS
IF (JFIN.EQ.0) ALX=ALB
425 CONTINUE
IF (CPMN - CPN) 426,427,427
426 CONTINUE
CPMN     = CPN

```

```

PRS 038
PRS 039
PRS 040
PRS 041
PRS 042
PRS 043
PRS 044
PRS 045
PRS 046
PRS 047
PRS 048
PRS 049
PRS 050
PRS 051
PRS 052
PRS 053
PRS 054
PRS 055
PRS 056
PRS 057
PRS 058
PRS 059
PRS 060
PRS 061
PRS 062
PRS 063
PRS 064
PRS 065
PRS 066
PRS 067
PRS 068
PRS 069
PRS 070
PRS 071
PRS 072
PRS 073
PRS 074
PRS 075
PRS 076

```

```

ALN      = (XODB - XOD(LT))*DBAS
IF (JFIN.EQ.0) ALN=ALB
427 CONTINUE
IF (JFIN) 429,429,433
429 CONTINUE
PDESMX   = FS*(CPMX*QB - DELTAP)
PDESMN   = FS*(CPMN*QB - DELTAP)
RETURN
END
$IBFTC C53S9 DECK
SUBROUTINE INSTBL
C
C      THIS SUBROUTINE PERFORMS A GENERAL INSTABILITY ANALYSIS
C      FOR SANDWICH CYLINDERS. THE METHOD, DEVELOPED BY BO ALMROTH
C      OF LMSC SOLID MECHANICS LABORATORY, HANDLES AXIAL COMPRESSION
C      PLUS LATERAL PRESSURE. THE ANALYSIS IS ELASTIC, AND INCLUDES
C      TRANSVERSE SHEAR EFFECTS. A KNOCKDOWN FACTOR BASED ON (R/T)
C      EFFECTIVE IS APPLIED TO THE CLASSICAL LOAD. LATERAL PRESSURE
C      MAY BE EITHER BURSTING OR CRUSHING.
C
C      DIMENSION X(44),A2(2,44),Z(4)
C      COMMON/KLSS/XL
C      COMMON/INSTB/ E , G1, G2, XNU, T1, T2, RD, HT, CRG, K,P,KG
C      COMMON/KLASS/ CRW, XNCL, PBAR, XN, XN1, XN2, XN11, XN12, XN22
C      COMMON/INTRP/ X, A2, Z, XNMIN
C      COMMON/SFCTN/ GBAR, F
101 FORMAT(6E12.8)
201 FORMAT(1H0,49HCASE SKIPPED SINCE GBAR WAS OUTSIDE RANGE - GBAR=E11.4)
1,4)
202 FORMAT(1H0,43HCASE SKIPPED SINCE F WAS OUTSIDE RANGE - F=E11.4)
210 FORMAT(1H0,5HXNCR=E15.8)
KG      = 0
Z(1) = 1.E-02
Z(2)=1.E-03
Z(3)=1.E-04
Z(4)=1.E-05
E1=E
E2=E
TR=T1/T2
R=RD/HT

```

```

H=HT/T2
89 F=(TR**3+E2/E1)*(TR+1.0)**2/(H*H*TR*(1.0+E2/E1*TR))/12.0
   GBAR=0.5*E1/G1*SQRT(E2/E1*TR/(1.0-XNU**2))/(R*H)*(1.0-(TR+1.0)/(2.0*H))
10*H)
   IF(GBAR-X(1))52,10,10
52 XNMIN=0.123
   GO TO 531
10 IF(GBAR-X(44))11,11,9
9 CONTINUE
   KG      = 1
   GO TO 500
11 IF(F-Z(4))12,13,13
12 F = Z(4)
13 IF (F - Z(1)) 51, 51, 122
122 F = Z(1)
51 CALL INTERP
531 CALL CLASS
   K=1
53 XNR=2.0*XNCL*E1*SQRT(E2/E1*TR/(1.0-XNU**2))/(H*R*R)
   XNN=XNMIN/XNCL
      XMIN IS BASED ON G1 = G2 (SQUARECELL CORE). XMIN ALSO
      NEGLECTS PRESSURE EFFECTS. HOWEVER, THE KNOCKDOWN FACTOR IS
      APPROXIMATE, SO XMIN WILL BE ADEQUATE FOR OLTHER G1/G2
      VALUES AND FOR LOW PRESSURES.
   IF(XNN-1.0)15,15,14
14 XNN=1.0
15 RTE=R*H*SQRT(TR+E2/E1)/SQRT(TR**3+E2/E1+12.0*H*H*TR*(1.0+E2/E1*TR)
   1/((TR+1.0)**2))
   IF (RTE .GT. 33.0) GO TO 17
16 PHI=1.0
   GO TO 18
17 PHI=6.48/(RTE**0.54)
18 C=(PHI-0.12)/0.88
   XNXR=XNR*(XNN+C*(1.0-XNN))
   CRG=XNXR*RD
49 CONTINUE
500 CONTINUE
   RETURN
   END

```

```

$1BFTC C53S10  DECK
SUBROUTINE INTERP
C      THIS SUBROUTINE IS UTILIZED BY INSTBL
      DIMENSION X(44),A2(2,44),Z(4)
      COMMON/INTRP/ X, A2, Z, XNMIN
      COMMON/SFCTN/ GBAR, F
      DIVDI(Q000FL,Q001FL,Q002FL,Q003FL,Q004FL,Q005FL,Q006FL)=Q001FL+(Q0TRP
106FL-Q000FL)/(Q002FL-Q000FL)*(Q003FL-Q001FL)+(Q006FL-Q000FL)*(Q006TRP
2FL-Q002FL)/(Q004FL-Q000FL)*(Q005FL-Q003FL)/(Q004FL-Q002FL)-(Q003FTRP
3L-Q001FL)/(Q002FL-Q000FL))
      ABSLOG(Q007FL,Q008FL,Q009FL,Q010FL,Q011FL)=Q008FL+(ALOG(Q011FL)-ALTRP
10G(Q007FL))/(ALOG(Q009FL)-ALOG(Q007FL))*(Q010FL-Q008FL)
      M=0
      N=0
      IT=-1
      DO 9 I=1,44
      IF(GBAR-X(I))8,10,9
          9 CONTINUE
10 M=1
      8 DO 11 J=1,4
      IF(F-Z(J))11,13,12
          11 CONTINUE
13 N=J
12 IF(GBAR-0.2)14,14,15
14 IF(F-Z(2))16,16,17
16 K=2
      GO TO 25
17 K=1
25 IF(M)18,18,19
19 XNMIN=A2(K,M)
      GO TO 24
18 IF(I-2)20,20,21
20 I=I+1
21 X0=X(I-2)
      X1=X(I-1)
      X2=X(I)
      XC=GBAR
27 IF(K-2)26,26,28

```

```

28 K=2
26 Y0=A2(K,I-2)
   Y1=A2(K,I-1)
   Y2=A2(K,I)
   XNMIN=DIVDI(X0,Y0,X1,Y1,X2,Y2,XC)
   IF(IT)24,29,30
24 IF(K-1)22,22,99
22 IF(N)23,23,99
23 IF(I-2)4,4,5
4 I=I+1
5 X0=X(I-2)
  X1=X(I-1)
  X2=X(I)
  XC=GBAR
  U0=Z(1)
  U1=Z(2)
  U=F
  Y0=A2(2,I-2)
  Y1=A2(2,I-1)
  Y2=A2(2,I)
  V1=DIVDI(X0,Y0,X1,Y1,X2,Y2,XC)
  V0=XNMIN
  XNMIN=ABSLOG(U0,V0,U1,V1,U)
  GO TO 99
15 IF(N)33,33,34
34 K=N
   GO TO 18
33 K=J-1
   IT=0
   GO TO 18
29 DO=XNMIN
  K=J
  IT=1
   GO TO 18
30 D1=XNMIN
  B0=Z(J-1)
  B1=Z(J)
   IF(D0-D1)32,99,32
32 XNMIN=ABSLOG(B0,D0,B1,D1,F)

```

```

TRP 039
TRP 040
TRP 041
TRP 042
TRP 043
TRP 044
TRP 045
TRP 046
TRP 047
TRP 048
TRP 049
TRP 050
TRP 051
TRP 052
TRP 053
TRP 054
TRP 055
TRP 056
TRP 057
TRP 058
TRP 059
TRP 060
TRP 061
TRP 062
TRP 063
TRP 064
TRP 065
TRP 066
TRP 067
TRP 068
TRP 069
TRP 070
TRP 071
TRP 072
TRP 073
TRP 074
TRP 075
TRP 076
TRP 077

```

```

99 RETURN
END
$IBFTC C53S11 DECK
SUBROUTINE CLASS
NON-SYMMETRIC BUCKLING PROGRAM. F. BROGAN. B. ALMROTH, B. BURNS
WHEN K=0, SUBROUTINE CLASS PICKS INITIAL ESTIMATES FOR THE
INDEPENDENT VARIABLES.
IF K GREATER THAN 0, PREVIOUS SOLUTIONS ARE USED FOR STARTING
ESTIMATES FOR THE NEXT CASE.
IF IOUT=0, NO INTERMEDIATE OUTPUT WILL BE PRINTED.
SET IOUT=1 IN SUBROUTINE CLASS IF INTERMEDIATE OUTPUT IS DESIRED.
COMMON/KLSS/XL
COMMON/KLASS/ CRW, XNCL, PBAR, XN, XN1, XN2, XN11, XN12, XN22
COMMON/INSTB/ EX, G1, G2, ZNU, T1, T2, RD, HT, CRG, K, P
COMMON/SFCTN/ GBAR, F
901 FORMAT ( 48H0 THE VALUES OF N, N1, N2, Z1, Z2, B, AND E ARE /
1 7E16.8 )
902 FORMAT ( 59H0 THE NON-SYMMETRIC BUCKLING PROGRAM HAS NOT CONVERGED
1 IN 14, 12H ITERATIONS. / 3H B= E16.8, 2HE= E16.8 )
903 FORMAT ( 48H0 THE NON-SYMMETRIC BUCKLING HAS CONVERGED IN 14,
1 11H ITERATIONS / 33H THE VALUES OF N, B, AND E ARE / 3E16.8)
P12=3.1415927**2
ZN1=SQRT (1.-ZNU**2)
AP=P12*ZN1*RD*HT/XL**2
KBR=AMAX1(KBR,1)
IBR=KBR
CFT=.75
IOUT=0
DEL=.1
DLTA=.0001
EH=EX*HT
ZMU=G1/G2
DELX=DEL
G=GBAR
M1=75
M2=75
M4=40
PBAR=P/EH*RD**2*SQRT ((1.-ZNU**2)/(T1*T2))

```

```

TRP 078
TRP 079
CLS 001
CLS 002
CLS 003
CLS 004
CLS 005
CLS 006
CLS 007
CLS 008
CLS 009
CLS 010
CLS 011
CLS 012
CLS 013
CLS 014
CLS 015
CLS 016
CLS 017
CLS 018
CLS 019
CLS 020
CLS 021
CLS 022
CLS 023
CLS 024
CLS 025
CLS 026
CLS 027
CLS 028
CLS 029
CLS 030
CLS 031
CLS 032
CLS 033
CLS 034
CLS 035
CLS 036

```

```

PS=PBAR
KM=K
ITER=0
IF (K) 101, 101, 104
101 IF (G-.5) 102, 103, 103
102 ZI=1./(2.-4.*G)
GO TO 104
103 ZI=1./(2.*SQRT (F))
104 CONTINUE
DX=.001*ZI
DX2=2.*DX
DXX=DX*DX
105 CONTINUE
F1=SFUN(ZI-DX)
F2=SFUN(ZI)
F3=SFUN(ZI+DX)
DZ=(F3-F1)/DX2*DXX/(F3-2.*F2+F1)
ZI=ZI-DZ
IF (ABS (DZ/ZI)-DLTA) 120, 120, 110
110 ITER=ITER+1
IF (ITER-M1) 105, 105, 130
120 AN=F2
IF (IOUT) 150, 150, 121
121 WRITE (6,908)ITER, F2, ZI
GO TO 150
130 WRITE (6,906)ITER
AN=1.
906 FORMAT (40H0 THE SYMMETRIC BUCKLING HAS FAILED IN 14,
1 12H ITERATIONS )
908 FORMAT (10H0 ITER= 14, 5H AN= E16.7, 4H Z= E16.8)
150 CONTINUE
ITER=0
IT2=0
B=BS
E=ES
IF (K) 3, 3, 4
3 B=1.
E=1.
KBR=1

```

CLS 037
 CLS 038
 CLS 039
 CLS 040
 CLS 041
 CLS 042
 CLS 043
 CLS 044
 CLS 045
 CLS 046
 CLS 047
 CLS 048
 CLS 049
 CLS 050
 CLS 051
 CLS 052
 CLS 053
 CLS 054
 CLS 055
 CLS 056
 CLS 057
 CLS 058
 CLS 059
 CLS 060
 CLS 061
 CLS 062
 CLS 063
 CLS 064
 CLS 065
 CLS 066
 CLS 067
 CLS 068
 CLS 069
 CLS 070
 CLS 071
 CLS 072
 CLS 073
 CLS 074
 CLS 075

```

IBR=1
Z2=0.
PBAR=0.
BS=B
ES=E
IF (AP-1.) 4, 4, 7
7 E=2.*AP
ES=E
4 CONTINUE
Y=1.-ZNU
Y1=Y+1.
Y2= (1.+F)/4.
Y3=.25*G
Y4=2.*Y*ZMU
Y5=2.*ZMU*ZNU
A=1./(G*E)
A1=A+Y
BB=B*B
V=1.+BB
W=2.*B
V1=Y*ZMU*BB
V2=A+V1
W2=Y*ZMU*W
V3=A+Y+BB
W3=W
V4=V2+ZMU
W4=W2
V5=V3*V4-ZMU*BB
W5=W3*V4+V3*W4-ZMU*W
V11=1./BB
W11=-W*V11**2
V14=1./V5
W14=-W5*V14**2
V6=V/V5
W6=V*W14+W*V14
V7=V2*V6
W7=W2*V6+V2*W6
V9=V*V/BB

```

CLS 076
CLS 077
CLS 078
CLS 079
CLS 080
CLS 081
CLS 082
CLS 083
CLS 084
CLS 085
CLS 086
CLS 087
CLS 088
CLS 089
CLS 090
CLS 091
CLS 092
CLS 093
CLS 094
CLS 095
CLS 096
CLS 097
CLS 098
CLS 099
CLS 100
CLS 101
CLS 102
CLS 103
CLS 104
CLS 105
CLS 106
CLS 107
CLS 108
CLS 109
CLS 110
CLS 111
CLS 112
CLS 113

W9=W+W11	CLS 114
V15=1./V9	CLS 115
W15=-W9*V15**2	CLS 116
B2=-B*V7	CLS 117
B2B=-V7-B*W7	CLS 118
C2=-A1*V6	CLS 119
C2B=-A1*W6	CLS 120
V12=Y3*(B2**2+ZMU*C2**2)	CLS 121
W12=2.*Y3*(B2*B2B+ZMU*C2*C2B)	CLS 122
V20=A+4.*BB	CLS 123
W20=8.*B	CLS 124
V21=1./V2	CLS 125
V23=V21**2	CLS 126
W21=-W20*V23	CLS 127
V22=8.*B*BB	CLS 128
W22=24.*BB	CLS 129
B1=-V21*V22	CLS 130
B12=-V22*W21-W22/V20	CLS 131
V24=ZMU*C2	CLS 132
W24=ZMU*C2B	CLS 133
B1B=B1**2	CLS 134
V25=BB*B1B	CLS 135
V26=B1*B12	CLS 136
W25=2.*(B*B1B+BB*V26)	CLS 137
V30=B2+B*V24	CLS 138
W30=B2B+V24+B*W24	CLS 139
V31=B*(Y1+BB)	CLS 140
W31=Y1+3.*BB	CLS 141
V32=B2*V31	CLS 142
W32=B2*W31+B2B*V31	CLS 143
V33=ZMU*(1.+Y1*BB)	CLS 144
W33=Y1*ZMU*W	CLS 145
V34=C2*V33	CLS 146
W34=C2*W33+C2B*V33	CLS 147
V35=2.*Y*V30**2	CLS 148
W35=4.*Y*W30*V30	CLS 149
V36=B2*C2	CLS 150
W36=B2*C2B+B2B*C2	CLS 151
V37=Y5*BB*V36	CLS 152

```

W37=Y5*(V36+B*W36)
S2=2.*(V25+V24**2+V37+V35)+4.*(V32+V34)
S2B=2.*(W25+2.*V24*W24+W37+W35)+4.*(W32+W34)
V16=V12+E*S2/8.-PBAR
W16=W12+E*S2B/8.
AE=-G*A*A
U2=AE
U5=AE*(V3+V4)
U14=-U5*V14**2
U6=V*U14
U7=U2*V6+V2*U6
B2E=-B*U7
U21=-AE*V23
B1E=-V22*U21
C2E=-A1*U6-AE*V6
U24=ZMU*C2E
U12=2.*Y3*(B2*B2E+V24*C2E)
U25=2.*B8*B1*B1E
U30=B2E+B*U24
U32=B2E*V31
U34=C2E*V33
U35=4.*Y*V30*U30
U36=B2*C2E+B2E*C2
U37=Y5*B*U36
S2E=2.*(U25+2.*V24*U24+U37+U35)+4.*(U32+U34)
U16=U12+S2/8.+E*S2E/8.
XN=Y2*E*V9+V15/E+V11*V16
EB=AP*W11
XN1=Y2*E*W9+W15/E+V11*W16+W11*V16
XN2=Y2*V9+G*V15*AE+V11*U16
FN1=XN1+XN2*EB
IF (KM) 34, 34, 48
34 IF (ITER) 40, 40, 35
35 IF (XN-XNA) 40, 45, 45
40 CONTINUE
XNAB=SQRT (XN1**2+XN2**2)/DELX
XNA=XN
IT2=IT2+1

```

CLS 153
CLS 154
CLS 155
CLS 156
CLS 157
CLS 158
CLS 159
CLS 160
CLS 161
CLS 162
CLS 163
CLS 164
CLS 165
CLS 166
CLS 167
CLS 168
CLS 169
CLS 170
CLS 171
CLS 172
CLS 173
CLS 174
CLS 175
CLS 176
CLS 177
CLS 178
CLS 179
CLS 180
CLS 181
CLS 182
CLS 183
CLS 184
CLS 185
CLS 186
CLS 187
CLS 188
CLS 189
CLS 190

```

GO TO (36, 37), IBR
36 CONTINUE
  Z1=XN1/XNAB
  Z2=XN2/XNAB
  GO TO 39
37 CONTINUE
  Z1=DELX*FN1/ABS (FN1)
  Z2=0.
39 CONTINUE
  IF (IT2-M4) 47, 47, 44
44 Z1=CFT*Z1
  Z2=CFT*Z2
  DELX=CFT*DELX
  M4=15
  IT2=0
  GO TO 47
45 IF (DELX-.1*DLTA) 60, 60, 46
46 DELX=DELX/3.
  B=B+.66667*Z1
  E=E+.66667*Z2
  Z1=Z1/3.
  Z2=Z2/3.
  M4=10
  IT2=0
  GO TO 49
48 CONTINUE
  WB2=Y4
  WB5=2.*(W3*W4+V4+Y*ZMU*V3-ZMU)
  WB11=6.*V11**2
  WB14=-2.*W5*V14*W14-WB5*V14**2
  WB6=2.*(W*W14+V14)+V*WB14
  WB7=2.*W2*W6+WB2*V6+V2*WB6
  WB9=2.+WB11
  WB15=-WB9*V15**2+2.*W9**2*V15**3
  B2BB=-2.*W7-B*WB7
  C2BB=-A1*WB6
  WB12=2.*Y3*(B2*B2BB+B2B**2+ZMU*(C2*C2BB+C2B**2))
  WB21=-2.*W20*V21*W21-8.*V23
  B12B=-2.*(W22*W21)-V22*WB21-48.*B/V20

```

CLS 191
 CLS 192
 CLS 193
 CLS 194
 CLS 195
 CLS 196
 CLS 197
 CLS 198
 CLS 199
 CLS 200
 CLS 201
 CLS 202
 CLS 203
 CLS 204
 CLS 205
 CLS 206
 CLS 207
 CLS 208
 CLS 209
 CLS 210
 CLS 211
 CLS 212
 CLS 213
 CLS 214
 CLS 215
 CLS 216
 CLS 217
 CLS 218
 CLS 219
 CLS 220
 CLS 221
 CLS 222
 CLS 223
 CLS 224
 CLS 225
 CLS 226
 CLS 227
 CLS 228
 CLS 229

WB24=ZMU*C2BB	CLS 230
WB25=2.*(B1B+4.*V26*B+BB*(B12**2+B1*B12B))	CLS 231
WB30=B2BB+2.*W24+ZMU*B*C2BB	CLS 232
WB31=6.*B	CLS 233
WB32=2.*B2B*W31+B2*WB31+B2BB*V31	CLS 234
WB34=2.*C2B*W33+C2*2.*Y1*ZMU+C2BB*V33	CLS 235
WB35=4.*Y*(W30**2+WB30*V30)	CLS 236
WB36=2.*B2B*C2B+B2*C2BB+B2BB*C2	CLS 237
WB37=Y5*(2.*W36+B*WB36)	CLS 238
S2BB=2.*(WB25+2.*W24**2+2.*V24*WB24+WB37+WB35)+4.*(WB32+WB34)	CLS 239
WB16=WB12+E*S2BB/8.	CLS 240
AEE=-2.*G*AE*A	CLS 241
USE=2.*AE**2+AEE*(V3+V4)	CLS 242
USB=AE*(W3+W4)	CLS 243
U14E=-2.*U5*V14*U14-U5E*V14**2	CLS 244
W14E=-U5B*V14**2-2.*U5*V14*W14	CLS 245
U6E=V*U14E	CLS 246
U6B=V*W14E+W*U14	CLS 247
U7E=2.*U2*U6+AEE*V6+V2*U6E	CLS 248
U7B=U2*W6+W2*U6+V2*U6B	CLS 249
B2EE=-B*U7E	CLS 250
B2EB=-U7-B*U7B	CLS 251
U21E=-2.*AE*V21*U21-AEE*V23	CLS 252
U21B=-2.*AE*V21*W21	CLS 253
B1EE=-V22*U21E	CLS 254
B1EB=-W22*U21-V22*U21B	CLS 255
C2EE=-A1*U6E-AEE*V6-2.*AE*U6	CLS 256
C2EB=-A1*U6B-AE*W6	CLS 257
U24E=ZMU*C2EE	CLS 258
U24B=ZMU*C2EB	CLS 259
U12E=2.*Y3*(B2E**2+B2*B2EE+U24*C2E+V24*C2EE)	CLS 260
U12B=2.*Y3*(B2*B2EB+B2B*B2E+W24*C2E+V24*C2EB)	CLS 261
U25E=2.*BB*(B1E**2+B1*B1EE)	CLS 262
U25B=4.*B*B1*B1E+2.*BB*(B12*B1E+B1*B1EB)	CLS 263
U30E=B2EE+B*U24E	CLS 264
U30B=B2EB+U24+B*U24B	CLS 265
U32E=B2EE*V31	CLS 266
U32B=B2EB*V31+B2E*W31	CLS 267

```

U34E=C2EE*V33
U34B=C2E*W33+C2EB*V33
U35E=4.*Y*(U30**2+V30*U30E)
U35B=4.*Y*(V30*U30B+W30*U30)
U36E=2.*B2E*C2E+B2*C2EE+B2EE*C2
U36B=B2B*C2E+B2*C2EB+B2EB*C2+B2E*C2B
U37E=Y5*B*U36E
U37B=Y5*U36+Y5*B*U36B
S2EE=2.*(U25E+2.*(U24**2+V24*U24E)+U37E+U35E)+4.*(U32E+U34E)
S2EB=2.*(U25B+2.*(W24*U24+V24*U24B)+U37B+U35B)+4.*(U32B+U34B)
U16E=U12E+S2E/4.+E*S2EE/8.
U16B=U12B+S2B/8.+E*S2EB/8.
EBB=AP*WB11
EB2=EB**2
XN11=Y2*E*WB9+WB15/E+2.*W11*W16+V11*WB16+WB11*V16
XN12=Y2*W9+G*W15*AE+V11*U16B+W11*U16
XN22=G*V15*AE+ V11*U16E
GO TO (41, 42), IBR
41 CONTINUE
XK=XN12/XN11
Z2=(XN2-XK*XN1)/(XN22-XK*XN12)
Z1=(XN1-XN12*Z2)/XN11
GO TO 47
42 CONTINUE
FN11=XN11+2.*XN12*EB+XN22*EB2+XN2*EBB
Z1=FN1/FN11
Z2=0.
47 CONTINUE
B=B-Z1
E=E-Z2
49 GO TO (55, 57), IBR
55 IF (E-AP/B**2) 56, 56, 58
56 IBR=2
KBR=2
57 E=AP/B**2
58 CONTINUE
D1=ABS (Z1/B)
D2=ABS (Z2/E)
IF (IOUT) 53, 53, 51

```

CLS 268
CLS 269
CLS 270
CLS 271
CLS 272
CLS 273
CLS 274
CLS 275
CLS 276
CLS 277
CLS 278
CLS 279
CLS 280
CLS 281
CLS 282
CLS 283
CLS 284
CLS 285
CLS 286
CLS 287
CLS 288
CLS 289
CLS 290
CLS 291
CLS 292
CLS 293
CLS 294
CLS 295
CLS 296
CLS 297
CLS 298
CLS 299
CLS 300
CLS 301
CLS 302
CLS 303
CLS 304
CLS 305
CLS 306

```

51 CONTINUE
  WRITE (6,901) XN, XN1, XN2, Z1, Z2, B, E
53 CONTINUE
  IF (D1-DLTA) 50, 50, 52
50 IF ( D2-DLTA) 60, 60, 52
52 ITER=ITER+1
  IF (ITER-M2) 4, 4, 96
96 IF (KM) 97, 97, 98
97 KM=1
  PBAR=PS
  ITER=0
  GO TO 4
60 WRITE (6,903) ITER, XN, B, E
  IF (KM) 97, 97, 68
68 GO TO (61, 62), IBR
61 IBR=2
  ES=E
  BS=B
  XNS=XN
  B=1.
  ITER=0
  GO TO 7
62 GO TO (63, 65), KBR
63 IF (XN-XNS) 66, 64, 64
64 XN=XNS
  GO TO 66
65 CONTINUE
  BS=B
  ES=E
66 CONTINUE
  XNCL=AMIN1(AN,XN)
  XNCL=MIN1 (AN,XN)
  IF (KM) 97, 97, 99
98 WRITE (6, 902) ITER, B, E
  XNCL=AN
99 RETURN
  END
$IBFTC C53S12 DECK

```

CLS 307
 CLS 308
 CLS 309
 CLS 310
 CLS 311
 CLS 312
 CLS 313
 CLS 314
 CLS 315
 CLS 316
 CLS 317
 CLS 318
 CLS 319
 CLS 320
 CLS 321
 CLS 322
 CLS 323
 CLS 324
 CLS 325
 CLS 326
 CLS 327
 CLS 328
 CLS 329
 CLS 330
 CLS 331
 CLS 332
 CLS 333
 CLS 334
 CLS 335
 CLS 336
 CLS 337
 CLS 338
 CLS 339
 CLS 340
 CLS 341
 CLS 342
 CLS 343
 SFN 001

```

FUNCTION SFUN (Z)
DIMENSION X(44),A2(2,44),Z(4)
COMMON/SFCTN/ GBAR, F
ZX=4.*Z
SFUN=(1.+F)*Z+1./ZX-ZX*Z/(1./GBAR+ZX)
RETURN
END
$IBFTC C53S13 DECK
SUBROUTINE IREQ
COMMON/CONFG/THTA(10),ALF(10),NFMX,DBAS,DMN(10),I CONFG
COMMON/AERLOD/ALPHA,AMACH,NF
COMMON /CHKLD/ C1,C3,C4,ALB,DELTA,CHK,RAXMIN,RAXMAX,RPMIN,RPMAX,
1 CHKWND,CHKLEE,C2
COMMON ANFIMN,ANFIMX,DSUBB,PDESMN,PDESMX,E
COMMON/IRQ/ TTM, PI, ALONE, C6, AIREQ
IF (THTA(NF).LT. 0.01) GO TO 990
AIREQ = C6*ALB *((DSUBB - C2*ALB )**2)*(PDESMX**1.33333)
1 /TTM**1.33333
GO TO 994
C
C DETERMINE RING MOMENT OF INERTIA REQUIREMENTS TO PREVENT
C GENERAL INSTABILITY BY THE METHOD OF SHANLEY. SEE BECKER'S
C HANDBOOK OF STRUCTURAL STABILITY, VOL. 6, P24. (NACA TN 3786)
990 AIREQ = (ALB * (2.5E - 04) * PI * ANFIMX * (DSUBB / 2.) ** 4)/
1 (E * ALONE)
994 CONTINUE
RETURN
END
$IBFTC C53S14 DECK
SUBROUTINE RING
COMMON/RNG/ IKIND, AOT, BOT, COT, TSTD(30), K, AIRING, DD
COMMON /RSTRSS/ AA, TF, Z, CTH, AST, AISTT, FRING
C
C RINGS ARE ASSUMED TO BE FORMED SHEET METAL OF STANDARD
C GAUGE. IKIND INDICATES THE TYPE OF RING...1 = ANGLE, 2 = ZEE,
C 3 = HAT. BASE FLANGE WIDTH IS GIVEN BY B, RING DEPTH IS A,
C AND OUTSTANDING OR UPPER FLANGE WIDTH IS GIVEN BY C. SHEET
C THICKNESS IS INDICATED BY TT.
TT = TSTD(K)
A = AOT * TT
B = BOT * TT

```

```

C
GO TO (1020,1030,1040), IKIND
1020 AST = COT * TT
DD = B*TT + (A-TT) * TT
AIB = ( A*A*TT/2. + (B-TT)*(TT/2.)*2 )/AST
AIF = ((B-TT) * TT**3)/12.
AB = (A**3)*TT/12.
AF = (B-TT)*TT
= A*TT
DB = DD - TT/2.
DF = A/2. -DD
AIST=AIB+AIF+AB*DB+AF*DF*DF
GO TO 105
1030 AST = A*TT + 2.*(B-TT)*TT
DD = A/2.
AIB = (B-TT) * (TT**3)/12.
AIW = TT*(A**3)/12.
DB = (A-TT)/2.
AB = (B-TT)*TT
AIST=AIB*2.+AIW+AB*DB*DB*2.
GO TO 105
1040 AST=2.*B*TT +C*TT + 2.*(A-2.*TT)*TT
DD = ((B-TT)*TT*TT + A*A*TT + (C-2.*TT)*(A-TT/2.)*TT)/AST
AB = 2. * (B-TT)*TT
AW = 2.0 * A * TT
AT = (C -2.*TT)*TT
AIB = (B-TT)*(TT**3)/12.
AIW = TT*(A**3)/12.
AIT = (C-2.*TT)*(TT**3)/12.
DB = DD-TT/2.
DT = (A-TT/2. -DD)
DW = DD - A/2.
AIST=AIB*2.+AIW*2.+AIT+AB*2.*DB*DB
1+AW*DW*DW*2.+AT*DT*DT
1050 CONTINUE
ECC = DD + .5 *TF
Z = (AST * ECC) / (AST + B * TF)
CALCULATE MOMENT OF INERTIA OF RING,INCLUDING ONE FACE.
AISTT = AIST + (B * TF**3 / 12.) + AST * (ECC - Z)**2 +

```

RNG 013
 RNG 014
 RNG 015
 RNG 016
 RNG 017
 RNG 018
 RNG 019
 RNG 020
 RNG 021
 RNG 022
 RNG 023
 RNG 024
 RNG 025
 RNG 026
 RNG 027
 RNG 028
 RNG 029
 RNG 030
 RNG 031
 RNG 032
 RNG 033
 RNG 034
 RNG 035
 RNG 036
 RNG 037
 RNG 038
 RNG 039
 RNG 040
 RNG 041
 RNG 042
 RNG 043
 RNG 044
 RNG 045
 RNG 046
 RNG 047
 RNG 048
 RNG 049
 RNG 050

```

1  B * TF * Z**2
AA  = A
AIRING = AIST
RETURN
END
$IBFTC C53S15 DECK
BLOCK DATA
DIMENSION X(44),A2(2,44),Z(4)
COMMON/RNG/ IKIND, AOT, BOT, COT, TSTD(30), K, AIRING, AST, DD
COMMON/INTRP/ X, A2, Z, XNMIN
DATA TSTD/ .008, .012, .016, .020, .025, .032, .040, .050, .063,
1 .071, .080, .090, .100, .125, .190, .250, .312, .375, .438, .500,
2 .562, .625, .688, .750, .812, .875, .938, 1.000, 1.125, 1.250 /
DATA X/.003, .004, .005, .006, .007, .008, .009, .01, .02, .03, .04, .05, .06,
1 .07, .08, .09, .1, .2, .3, .4, .5, .6, .7, .8, .9, 1.1, 1.2, 1.3, 1.4, 1.5, 1.6, 1.7, 1.8,
2 9.10, 10.20, 30.40, 50.60, 70.80, 90.100, /
END
$IBFTC C53S16 DECK
SUBROUTINE RSTRES
COMMON ANFINM,ANFIMX,DSUBB,PDESMN,PDESMX,E
COMMON /CHKLD/ C1,C3,C4,ALB,DELTAS,CHK,RAXMIN,RAXMAX,RPMIN,RPMAX,
1  CHKWND,CHKLEE,C2
COMMON /RSTRSS/ AA, TF, Z, CTH, AST, AISTT, FRING
ZFLG = AA + .5 * TF - Z
ANTHTA = PDESMX * DSUBB / (2. * CTH)
ANRING = ANHTA * ALB * (AST / (AST + 2. * ALB * TF))
C
C  ESTIMATE COMPRESSIVE RING STRESS DUE TO OUT-OF-ROUNDNESS
AND ASYMMETRY OF LOADING. (F = PR)
FBEND = ANRING * ZFLG / AISTT
FRING = ANRING / AST + FBEND
RETURN
END
$IBFTC C53S17 DECK
SUBROUTINE TNOSST
C
C  THE FOLLOWING SUBROUTINE CALCULATES NOSE CAP SKIN THICK-TNS
C  NESS REQUIRED FOR STRUCTURAL INTEGRITY. THE FAILURE CRITERION
C  USED IS THAT PRESENTED FOR NON-SHALLOW SPHERICAL CAPS IN THE
C  LMSC STRUCTURAL METHODS HANDBOOK, SECTION 6.32.1, DATED
C  30 SEPTEMBER 1962.
TNS 001
TNS 002
TNS 003
TNS 004
TNS 005
TNS 006
TNS 007
RNG 051
RNG 052
RNG 053
RNG 054
RNG 055
BLK 001
BLK 002
BLK 003
BLK 004
BLK 005
BLK 006
BLK 007
BLK 008
BLK 009
BLK 010
BLK 011
BLK 012
RSS 001
RSS 002
RSS 003
RSS 004
RSS 005
RSS 006
RSS 007
RSS 008
RSS 009
RSS 010
RSS 011
RSS 012
RSS 013
RSS 014
RSS 015
TNS 001
TNS 002
TNS 003
TNS 004
TNS 005
TNS 006
TNS 007

```

```

COMMON/TNOS/TMINN,PDSPH,TCAPST,RCAP
COMMON ANFIN,ANFIMX,DSUBB,PDESMN,PDESX,DE
C      CONSTANT USED IN CALCULATING THE COLLAPSE PRESSURE FOR
C      THE NOSE CAP IS
C      =0.606*E/(RCAP**2)
C      SHELL THICKNESS IS SET EQUAL TO TMINN, AND COLLAPSE
C      PRESSURE OF THE NOSE CAP IS CALCULATED. IF COLLAPSE PRESSURE
C      IS LESS THAN THE DESIGN PRESSURE, SKIN THICKNESS IS INCREASED
C      BY INCREMENTS OF 0.001 INCH UNTIL COLLAPSE PRESSURE IS EQUAL
C      TO OR GREATER THAN THE DESIGN PRESSURE.
C      TCAPST = TMINN -0.001
90 TCAPST = TCAPST + 0.001
B      = 0.04*SQRT (RCAP/TCAPST)
PCOLL = A * (TCAPST ** 2)/EXP(B)
IF (PCOLL - PDSPH) 90,91,91
91 CONTINUE
RETURN
END
TNS 008
TNS 009
TNS 010
TNS 011
TNS 012
TNS 013
TNS 014
TNS 015
TNS 016
TNS 017
TNS 018
TNS 019
TNS 020
TNS 021
TNS 022
TNS 023
TNS 024
TNS 025

```

APPENDIX B
DEFINITIONS OF VARIABLE NAMES

A	-	The length of the free side of a honeycomb cell, in.
AA	-	The height of the stiffening ring, in.
AFACT	-	Used to determine which equations to use in determining wrinkling stress.
AGWRIN	-	Allowable face wrinkling stress, psi.
AIREQ	-	Moment of inertia required of the stiffening ring cross-section, in. ⁴
AIRING	-	Moment of inertia of the stiffening ring, in. ⁴
AISTT	-	Moment of inertia of ring and effective skin, in. ⁴
ALB	-	Bay length, in.
ALCAP	-	Axial length of nose cap, in.
ALCONE	-	Frustum length, in.
ALF (NF)	-	Length of frustum number NF, in.
ALMX2	-	Axial distance from base of bay to top of frustum, in.
ALPHA	-	Angle of attack, degrees when read in and radians when used in computations.
ALTOT	-	Total length of fairing, in.
AMACH	-	Mach number.
AMU	-	Poisson's ratio.
ANFLAX	-	Line load contributed by axial loading on bay, lbs/in.
ANFIB	-	Line load contributed by bending moments on bay, lbs/in.
ANFICR	-	Critical line load, lbs/in.
ANFIMN	-	Total line load on windward side of fairing multiplied by factor of safety, lbs/in.
ANFIMX	-	Total line load on leeward side of fairing multiplied by factor of safety, lbs/in.
AOT	-	The ratio A/t (see Figure 3).

APRC	-	An equivalent radius of curvature of the particular bay under consideration.
AST	-	Cross-sectional area of ring, sq. in.
AXLDCP	-	Axial load contributed by nose cap, lbs.
AXLOAD	-	Axial load at some specified location on fairing, lbs.
B	-	The length of the bonded side of a honeycomb cell, in.
BEND	-	Bending moment at some specified location on fairing, lb-in.
BETA	-	The acute angle formed by the sides of a honeycomb cell, radians.
BETADG	-	The acute angle formed by the sides of a honeycomb cell, degrees.
BNDCAP	-	Bending moment of the nose cap about its base, lbs/in.
BOT	-	The ratio of width of attaching ring flange to ring thickness (see Figure 3).
CASE	-	The number of a particular set of data in a sequence of runs.
CBETA	-	Cosine of the angle BETA.
CCA	-	Honeycomb core allowable compressive stress, psi.
CDCAP	-	Drag coefficient for the spherical nose cap with the base area of the nose cap as a reference area.
CELLWD	-	Calculated honeycomb cell size, in.
CELSIZ	-	Honeycomb cell size reduced to a standard size, in.
CNCAP	-	Normal force coefficient per radian angle of attack for the spherical nose cap using nose cap base area as a reference area, /radian.
COT	-	The ratio of the width of the cap of stiffener to ring thickness (see Figure 3).
CPA(LT)	-	The change in pressure coefficient on either the windward or leeward side of the fairing due to angle of attack at station LT. (See Figures 4 and 5.)
CPAA	-	Same as CPA(LT).

CPO(LT)	-	Pressure coefficient at zero angle of attack at station LT.
CPOO	-	Same as CPO(LT).
CRG	-	Critical line load for general panel instability, lbs/in.
CSA	-	Sandwich core allowable shear stress, psi.
CTA	-	Sandwich core allowable tensile stress, psi.
D (I)	-	Small diameter of the Ith bay, in.
DBAS	-	Base diameter of fairing, in.
DBASE	-	Base diameter of a frustum, in.
DBNSD	-	Inside base diameter of bay, in.
DD	-	Distance from the centroid of the stiffening ring to the inside bay wall, in.
DELTAP	-	Difference between internal and free-stream pressure, psi.
DELTAS	-	Bay skin thickness, in.
DINS	-	Inside upper diameter of bay, in.
DMIN	-	Small diameter of frustum, in.
DMN	-	Same as DMIN.
DOD(LT)	-	Ratio of local diameter to fairing base diameter at station LT.
DOVDB(NF)	-	Ratio of small diameter of frustum NF to base diameter of fairing.
DSUBB	-	Base diameter of bay, in.
DUSE1	-	Diameter of area useful for payload at the base of the bay. (See Line SWD 0507 of program listing in Appendix A.)
DUSE2	-	Diameter of area useful for payload at top of bay. (See Line SWD 0508 of program listing in Appendix A.)
E	-	Modulus of elasticity of structural material, in.
ECORE	-	Modulus of elasticity of core material, psi.
EFACE	-	Modulus of elasticity of face material, psi.

ELMIN(NF)	-	Minimum bay length for frustum NF, in.
ENFIMN (I)	-	ANFIMN for Ith bay, lbs/in.
ENFIMX (I)	-	ANFIMX for Ith bay, lbs/in.
ENSUBR	-	Radius conversion factor for conical frustums.
FCA	-	Sandwich face allowable compressive stress, psi.
FCFB	-	Ring outstanding flange allowable buckling stress, psi.
FFAX (I)	-	Face axial stress at base of Ith bay, psi.
FFB (I)	-	Face bursting stress in Ith bay, psi.
FFC (I)	-	Face compressive stress in Ith bay, psi.
FFDMP	-	Face dimpling stress, psi.
FFS (I)	-	Face shear stress due to aerodynamic lift, psi.
FFWR (I)	-	Face wrinkling stress, psi.
FRING	-	Maximum compressive stress in ring flange, psi.
FS	-	Factor of safety.
FSA	-	Sandwich face allowable shear stress, psi.
FSBZCP	-	Shear force contributed by nose cap, lbs.
FSUBZ	-	Shear force at a specified location on the fairing, lbs.
FTA	-	Sandwich face allowable tensile stress, psi.
FZ (I)	-	Shear force at the base of the Ith bay, lbs.
G	-	Modulus of shear, psi.
GCORE	-	Modulus of shear of core material, psi.
GC1	-	Effective shear modulus of the honeycomb cellular material in the ribbon direction, psi.
GC2	-	Effective shear modulus of the core in the direction perpendicular to the ribbons, psi.
GFACE	-	Modulus of shear of face material, psi.
G1	-	Effective shear modulus of the core material corrected for height to cell size ratio, in the ribbon direction, psi.

G2	-	Effective shear modulus corrected for height to cell size ratio, perpendicular to the ribbon direction, psi.
H	-	Distance between midplanes of facing sheets, in.
HUSE	-	Useful axial length of nose cap, in.
I	-	Index indicating bay. Numbering begins at base of fairing.
IKIND	-	Type of ring. 1 = angle, 2 = Zee, 3 = hat.
IMAX	-	Total number of bays in fairing.
IMX(NF)	-	Number of bays from bottom of fairing to top of frustum NF.
IW	-	Index indicating bay number within a frustum.
I1	-	Lower index used for writing output data.
I2	-	Upper index used for writing output data.
J	-	Index indicating parameter associated with face thickness T (J).
JF	-	Index of skin thickness which is optimum for a bay.
JG	-	Index used for determining minimum adequate ribbon thickness.
KEY	-	Input parameter indicating type of output desired. (See Section 3.0 of the TECHNICAL DISCUSSION.)
KG	-	Parameter generated by Subroutine INSTBL, indicating that GBAR has exceeded the range in which it is valid.
KTFSTD	-	An input parameter which tells if standard gauges are to be used for sandwich faces.
RTRSTD	-	An input parameter which tells if standard gauges are to be used for core ribbons.
KY	-	An optional control used to obtain faster convergence in Subroutine INSTBL.

LPFL	-	An integer used to control computation of lateral pressures.
LPRES	-	An integer indicating type of pressure profile to be read in. (See Section 3.0 of TECHNICAL DISCUSSION.)
LSTOP	-	An integer indicating that the last pressure profile data card has been read.
LT	-	Index that identifies a particular point on the fairing.
LTMAX	-	Total number of pressure profile data points.
MAT	-	An integer indicating the material to be used. (See Section 3.0 of the TECHNICAL DISCUSSION.)
MATF	-	An integer indicating the type of material to be used for sandwich faces.
NBAY	-	Number of bays in a frustum.
NF	-	Index indicating frustum number.
NFMAX	-	Total number of frustums in the fairing.
P	-	Pressure differential, equal to either PDESMN or PDESMX, as appropriate, psi.
PDESMN	-	Maximum pressure differential across the sandwich, multiplied by the factor of safety, on the leeward side of the bay, psi.
PDESMX	-	Maximum pressure differential across the sandwich, multiplied by the factor of safety, on the windward side of the fairing, psi.
PDIFF	-	Maximum burst pressure differential, psi.
PDSPH	-	Maximum pressure differential across the nose cap skin, multiplied by a factor of safety, psi.
PHI	-	Angle between the applied load and the direction of the honeycomb ribbon.

PSTAT	-	Free-stream pressure, psi.
QBAR	-	Dynamic pressure, lbs/sq. ft.
RCAP	-	Radius of spherical nose cap, in.
RHO	-	Material density, lbs./cu. in.
ROCORE	-	Core material density, lbs/cu. in.
RUSE	-	Radius of nose cap volume which is useful for payload, in.
SCAP	-	Surface area of spherical nose cap, sq. in.
SLOPT	-	Sum of bay lengths within a frustum, in.
SUMAL	-	Distance from the base of the fairing to the base of the bay, in.
TCAP	-	Thickness of nose cap skin, in.
TCAPST	-	Thickness of nose cap skin required to withstand pressure loads, in.
TCAPTH	-	Thickness of nose cap skin required to limit its temperature to the maximum specified, in.
TCONTH	-	Thickness of skin required on the top frustum to limit its temperature to the maximum specified, in.
TCORE(I)	-	Honeycomb core height in the Ith bay, in.
TF	-	Sandwich face thickness, in.
TFACE(I)	-	Face thickness in the Ith bay, in.
THETA	-	Frustum half angle, radians
THTA(NF)	-	Half angle of frustum NF, degrees.
TKCIL	-	Skin thickness required to carry the circumferential load, in.
TKHOOP	-	Skin thickness required for hoop stress due to internal pressure, in.

TKMCL	-	Skin thickness required to carry the meridional compressive load, in.
TKSAL	-	Skin thickness required for shear stress due to aerodynamic lift, in.
TMINC	-	Minimum skin thickness to be used in designing a frustum, in.
TMINN	-	Minimum skin thickness to be used in designing the nose cap, in.
TMNC(NF)	-	TMINC for frustum NF, in.
TMP	-	Maximum allowable temperature for both the top frustum and nose cap, °F.
TMPMAX	-	Same as TMP.
TRBMIN	-	Input value of minimum ribbon thickness, in.
TRBN(I)	-	Ribbon thickness for the Ith bay, in.
TRBSTD(JG)	-	Block data values to be used for ribbon thickness if KTRSTD is equal to 1.
TRIBN	-	Ribbon thickness, in.
TRING (I)	-	Thickness of ring material for optimized design of the Ith bay, in.
TSTD	-	Block data of standard sheet metal gauges, in.
T1	-	Thickness of outer face of sandwich, in.
T2	-	Thickness of inner face of sandwich, in.
VCAP	-	Volume of nose cap which is useful for payload. (See HUSE and RUSE), cu. ft.
VCORE	-	Volume enclosed between the two sandwich faces, cu. in.
VGROS	-	Gross volume of nose cap, cu. in.
VGROSS	-	Gross volume of fairing, cu. ft.

VSEG (J)	-	Volume of bay designed for a skin gauge of T (J), cu. in.
VTOT	-	Useful volume of fairing (see DUSE1, DUSE2, HUSE and RUSE), cu. ft.
VUSE	-	Useful volume of frustum (see DUSE1, and DUSE2), cu. ft.
W	-	Average length of a core cell wall, equal to $1/2$ (ATB), in.
WBOND (I)	-	Weight of adhesive bond material used to attach faces to core, lbs.
WCAP	-	Weight of nose cap, lbs.
WCONE	-	Frustum weight, lbs.
WFACE (I)	-	Weight of both sandwich faces of the Ith bay, lbs.
WRING (I)	-	Weight of ring for the Ith bay, lbs.
WSEG (I)	-	Weight of the Ith bay, lbs.
WSPLCE (I)	-	Weight of panel splices in the Ith bay, lbs.
WTCORE (I)	-	Weight of core material in the Ith bay, lbs.
WTDEX (I)	-	Weight of bay divided by volume of bay, lbs/cu. in.
WTINFC	-	Weight of inner sandwich face material in the Ith bay, lbs.
WTOT	-	Total fairing weight, lbs.
WTOTFC	-	Weight of outer face material in the Ith bay, lbs.
XBCAP	-	Distance from base of nose cap to center of lift pressure on the nose cap, in.
XL	-	Length of bay, in.
Y	-	Warpage correction factor for thick cores.

APPENDIX C
AERODYNAMIC PRESSURE COEFFICIENTS

When data for the fairing pressure profile is not input, the pressure coefficient at zero angle of attack, CPOO, and the maximum change in pressure coefficient due to angle of attack, CPAA, are computed for each conical frustum in Subroutine AERO. For the purpose of computing these parameters, each frustum is treated as a complete cone with an attached shock. Both CPOO and CPAA will then be uniform in the axial direction for each frustum. In order to construct the pressure profile as described in Section 1.4 and Figure 4, the values computed for CPOO and CPAA for the frustum are assigned to the stations at the ends of the frustum. That is

$$\begin{aligned} \text{CPO (LT)} &= \text{CPOO} \\ \text{CPO (LT+1)} &= \text{CPOO} \\ \text{CPA (LT)} &= \text{CPAA} \\ \text{CPA (LT+1)} &= \text{CPAA} \end{aligned}$$

In which LT is the station at the small diameter of the frustum, and (LT+1) is the station at the large diameter of the frustum.

Using the ground rules indicated above, CPOO can be readily determined through the use of equations developed by Simon and Walter in Reference 2, which agree within a few percent with data presented in Chart 6 of Reference 3 (NACA Report 1135). These equations have been programmed in Subroutine AERO and are used in computing CPOO for each frustum.

When flying at an angle of attack the pressure distribution in the circumferential direction varies with circumferential position. This circumferential pressure distribution is assumed to be sinusoidal (see Figure 5). Since each frustum is treated as a complete cone, the distribution in the axial direction is uniform. The pressure distribution over the entire frustum can now be described by equations specifying the circumferential pressure distribution. These equations, as illustrated in Figure 5, are

$$C_P = C_{PO} - \Delta C_P \sin \phi \quad (C1)$$

$$P = P_{AV} - \Delta P_{MAX} \sin \phi \quad (C2)$$

In which

$$C_{PO} = \text{CPOO}$$

$$\Delta C_P = \text{CPAA}$$

The normal force, ΔF_N , produced on an incremental length, ΔX , by this pressure distribution can be computed as follows:

$$\Delta F_N = \Delta X \int_0^{2\pi} (-P \sin \phi) \left(\frac{D}{2}\right) d\phi \quad (C3)$$

in which D is the diameter of the increment. When the expression for P (Equation C2) is substituted in Equation C3, and the integration performed

$$\Delta F_N = \frac{\pi}{2} D (\Delta X) (\Delta P_{MAX}) \quad (C4)$$

In which

$$\Delta P_{MAX} = (\Delta C_P) q$$

and

$$\Delta F_N = \frac{\pi}{2} D (\Delta X) (\Delta C_P) q \quad (C5)$$

The normal force on the same increment of cone can also be computed by using the normal force coefficient, C_N . For a complete cone the normal force, F_N , is expressed by

$$F_N = C_N q A \quad (C6)$$

in which A is the base area of the cone. The incremental normal force, ΔF_N , produced by a short length, ΔX , of this cone is

$$\Delta F_N = C_N q (\Delta A) \quad (C7)$$

in which ΔA is the surface area of the increment projected on the cone base, expressed by

$$\Delta A = \frac{\Delta D}{2} (\pi D) \quad (C8)$$

The change in diameter, ΔD , for a change in length, ΔX , is

$$\Delta D = 2 (\Delta X) \tan \theta \quad (C9)$$

in which θ is the half angle of the cone. Substituting Equations C8 and C9 into C7 yields the following expression for ΔF_N :

$$\Delta F_N = C_N q (\pi D) (\Delta X) \tan \theta \quad (C10)$$

When the two expressions for ΔF_N (Equations C5 and C10) are equated and solved for ΔC_P , the following expression is obtained.

$$\Delta C_P = 2 C_N \tan \theta \quad (C11)$$

In Chart 8 of Reference 3 $C_{N\alpha}$ is plotted as a function of cone half angle θ and Mach number. $C_{N\alpha}$ is defined as

$$C_{N\alpha} = \left(\frac{\partial C_N}{\partial \alpha} \right)_{\alpha=0} \quad (C12)$$

in which α is the angle of attack. For small angles of attack the following relationship is valid:

$$C_N = (C_{N\alpha}) \alpha \quad (C13)$$

For the study for which this computer program was developed maximum loads occur in the neighborhood of Mach 1.5. In this region $C_{N\alpha}$ is not a strong function of Mach number. Therefore, a plot was made of $C_{N\alpha}$ versus θ at Mach 1.5. The points fell on a straight line expressed by the following equation.

$$C_{N\alpha} = 2.03 - 1.2 \theta \quad (C14)$$

In Equation C14 both α and θ are expressed in radians. Substituting Equations C13 and C14 into C11 yields the following equation which is used to compute CPAA in Subroutine AERO ($CPAA = \Delta C_P$).

$$\Delta C_P = (2 \tan \theta) (2.03 - 1.2 \theta) \alpha \quad (C15)$$

In Equation C15, ΔC_P is the difference between the pressure coefficient on the windward side of the fairing and the pressure coefficient at the neutral position on the fairing.

APPENDIX D
BENDING MOMENTS, AXIAL LOADS AND SHEAR LOADS

In order to design a bay within the nose fairing structure, it is necessary to know the magnitude of the loads to which the bay is subjected. In addition to lateral pressure there are bending moments and axial loads which are used in computing line loads (force per running inch on the circumference) for the bay being analyzed. These computations are performed in Subroutine LOAD using the pressure profile data which was either computed in Subroutine AERO or input to the program.

The pressure profile data consists of a number of points connected by straight line segments as illustrated in Figure 4. In order to compute axial loads, shear loads and bending moments, it is necessary to compute the contribution of each of these pressure profile increments to the total load. In computing these incremental loads the point on the increment nearest the base of the fairing is used as a reference point.

First, an equation is derived to represent the shear force contributed by a pressure profile increment at its reference point. Nomenclature for this derivation is illustrated in Figure D1. The expression for ΔC_P as a function of X/D between locations $(X/D)_1$ and $(X/D)_2$ is as follows:

$$\Delta C_P = \Delta C_{P1} - \frac{\Delta C_{P1} - \Delta C_{P2}}{(X/D)_1 - (X/D)_2} [(X/D)_1 - (X/D)] \quad (D1)$$

At a specified location ΔC_P is the difference between the pressure coefficient on the windward side of the fairing and the pressure coefficient at the neutral position on the fairing (see Figure 5). The relationship between the variables in Equation D1 and Figure 4 are as follows:

$$\Delta C_{P1} = CPA (LT+1)$$

$$\Delta C_{P2} = CPA (LT)$$

$$(X/D)_1 = XOD (LT+1)$$

$$(X/D)_2 = XOD (LT)$$

Let

$$A_2 = \frac{\Delta C_{P1} - \Delta C_{P2}}{(X/D)_1 - (X/D)_2} \quad (D2)$$

$$A_1 = \Delta C_{P1} - A_2(X/D)_1 \quad (D3)$$

Then, combining Equations D1, D2 and D3

$$\Delta C_P = A_1 + A_2(X/D) \quad (D4)$$

In a similar manner the following expression can be obtained for the local diameter D .

$$D = A_{D1} + A_{D2}(X/D) \quad (D5)$$

In which

$$A_{D2} = \frac{D_1 - D_2}{(X/D)_1 - (X/D)_2}$$

$$A_{D1} = D_1 - A_{D2}(X/D)_1$$

In Appendix C an equation for the normal force, ΔF_N , for an increment of length, ΔX , was derived for a sinusoidal pressure distribution in the circumferential direction. This equation (Equation C5) is as follows:

$$\Delta F_N = \frac{\pi}{2} D (\Delta X) (\Delta C_P) q$$

in which q is dynamic pressure. Using Equation C5 the running load in the axial direction, w , is expressed as follows:

$$\begin{aligned} w &= \frac{\Delta F_N}{\Delta X} \\ &= \frac{\pi}{2} D (\Delta C_P) q \end{aligned} \quad (D6)$$

By combining Equations D4, D5 and D6, the following expression is obtained for w ,

$$w = \frac{\pi}{2} q \left[A_1 A_{D1} + A_1 A_{D2}(X/D) + A_2 A_{D1}(X/D) + A_2 A_{D2}(X/D)^2 \right] \quad (D7)$$

The shear force, v , at any point on the increment is

$$\begin{aligned} v &= \int_{X_1}^X -w dX \\ &= -D_{base} \int_{(X/D)_1}^{X/D} w d(X/D) \end{aligned} \quad (D8)$$

In which D_{base} is the base diameter of the fairing. When the integration is performed, the following expression is the result.

$$v = \frac{\pi}{2} q D_{base} \left\{ \beta_1 \left[X/D - (X/D)_2 \right] + \beta_2 \left[(X/D)^2 - (X/D)_2^2 \right] + \beta_3 \left[(X/D)^3 - (X/D)_2^3 \right] \right\} \quad (D9)$$

In which

$$\beta_1 = A_1 A_{D1}$$

$$\beta_2 = \frac{A_1 A_{D2} + A_2 A_{D1}}{2}$$

$$\beta_3 = \frac{A_2 A_{D2}}{3}$$

The incremental shear force, v_i , at the reference point of the i th increment (the point nearest the fairing base), due to aerodynamic pressure acting on the i th increment, is obtained by substituting $(X/D)_1$ for X/D in Equation D9.

The incremental bending moment, M_i , about the reference point of the i th increment is expressed as follows:

$$M_i = \int_{X_2}^{X_1} v \, dx$$

$$= D_{base} \int_{(X/D)_2}^{(X/D)_1} v \, d(X/D) \quad (D10)$$

By substituting Equation D9 into Equation D10 and performing the integration, the following expression is obtained.

$$\begin{aligned}
M_i = \frac{\pi}{2} q D_{\text{base}}^2 & \left\{ \frac{\beta_1}{2} \left[(X/D)_1^2 - (X/D)_2^2 \right] \right. \\
& - \beta_1 (X/D)_2 \left[(X/D)_1 - (X/D)_2 \right] + \frac{\beta_2}{3} \left[(X/D)_1^3 - (X/D)_2^3 \right] \\
& - \beta_2 (X/D)_2^2 \left[(X/D)_1 - (X/D)_2 \right] + \frac{\beta_3}{4} \left[(X/D)_1^4 - (X/D)_2^4 \right] \\
& \left. - \beta_3 (X/D)_2^3 \left[(X/D)_1 - (X/D)_2 \right] \right\} \quad (D11)
\end{aligned}$$

These incremental shear loads and bending moments are now used to compute the shear load, V_{base} , and bending moment, M_{base} , at the base of the fairing.

$$V_{\text{base}} = \sum_{i=1}^I v_i + \text{Contribution of nose cap} \quad (D12)$$

$$M_{\text{base}} = \sum_{i=1}^I (v_i L_i + m_i) + \text{Contribution of nose cap} \quad (D13)$$

in which I is the total number of increments and L_i is the distance from the base of the fairing to the reference point of the i th increment. The shear moment contribution of the nose cap are discussed below.

As design of the fairing moves from the base toward the nose cap, shear and moment contributed by each of the increments of pressure profile are subtracted from the total shear and bending moment. In moving from the reference point of the $(i-1)$ th increment to the reference point of the i th increment the shear and bending moments at the i th reference point are computed as follows:

$$V_i = V_{i-1} - v_{i-1} \quad (D14)$$

$$M_i = M_{i-1} - m_{i-1} - V_i x_{i-1} \quad (D15)$$

in which x is the length of the increment.

Usually the location of the base of a bay will not coincide with the beginning or end of a pressure profile increment. In this case the pressure profile increment is divided at the base of the bay and each part is treated as a complete increment.

Computation of axial loads is handled in much the same manner as computation of shear force. Using nomenclature similar to that used previously for the ΔC_P calculations (see Figure D1) the equation for the pressure coefficient within a pressure profile increment is

$$C_P = B_1 + B_2 D \quad (D16)$$

in which

$$B_1 = C_{P1} - \frac{C_{P1} - C_{P2}}{D_1 - D_2} D_1 \quad (D17)$$

$$B_2 = \frac{C_{P1} - C_{P2}}{D_1 - D_2} \quad (D18)$$

The incremental axial load is

$$\Delta F_{ax} = \int_{D_1}^{D_2} (q C_P - \Delta P) (\pi D) \frac{dD}{2} \quad (D19)$$

in which ΔP is the difference between fairing internal pressure and ambient pressure. When the expression for C_P is substituted into this equation and the integration is performed, the following equation is obtained.

$$\Delta F_{ax} = \frac{\pi}{2} \left[\frac{1}{2} (q B_1 - \Delta P) (D_1^2 - D_2^2) + \frac{1}{3} q B_2 (D_1^3 - D_2^3) \right] \quad (D20)$$

Total axial load at the base of the fairing is computed by summing up the incremental loads plus the drag contributed by the nose cap. As design of the fairing progresses from the base towards the nose cap, increments of axial load are subtracted in a manner similar to that employed in computing shear loads.

When the bending moment and axial load are known at the base of a bay, the circumferential line load can be computed. This is the load per unit length of circumference parallel to the surface of the skin. The axial load places a uniform compressive load on the circumference. The bending moment places a compressive load on the leeward side and a tensile load on the windward side. The line load due to the axial force is

$$(N\phi)_{AX} = \frac{F_{ax}}{\pi D \cos \theta} \quad (D21)$$

in which

$$\begin{aligned} F_{ax} &= \text{Axial force at the base of the bay} \\ D &= \text{Diameter at the base of the bay} \\ \theta &= \text{Semi-vertex angle of the bay} \end{aligned}$$

Using the assumption that the strain in the skin due to bending is proportional to the distance from the neutral plane, the maximum contribution of bending moment to the line load is computed by the following equation:

$$(N\phi)_{BEND} = \frac{M}{\frac{\pi}{4} D^2} \frac{1}{\cos \theta} \quad (D22)$$

When line load due to bending is superimposed on line load due to axial force the total becomes

$$(N\phi)_{WND} = (N\phi)_{AX} - (N\phi)_{BEND} \quad (D23)$$

$$(N\phi)_{LEE} = (N\phi)_{AX} + (N\phi)_{BEND} \quad (D24)$$

The subscript WND indicates windward side, and the subscript LEE indicates the leeward side.

The contribution of the nose cap to axial load, shear force and bending moment are computed by means of the nose cap drag coefficient, C_D , normal force coefficient per radian angle of attack, C_{N_α} , and $\bar{X}$, the distance from the base of the nose cap to center of pressure on the normal plane. The reference area for C_D and C_{N_α} is the base area of the nose cap. These parameters can be read into the computer or computed in Subroutine LOAD.

The computations for the nose cap C_D in Subroutine LOAD are based on a computed pressure coefficient at the stagnation point, $(C_P)_{stg}$, and a pressure distribution over the nose cap described by the following equation:

$$C_P = (C_P)_{stg} \sin^2 \phi \quad (D25)$$

in which ϕ is the angle between a plane tangent to the nose cap surface and the line of flight.

In order to compute $(C_P)_{stg}$, the pressure at the stagnation point is assumed to be equal to the stagnation pressure downstream from a normal shock with upstream Mach number equal to that of the vehicle. For one-dimensional flow of a perfect gas with constant specific heat and molecular weight the ratio of downstream stagnation pressure, P_o , to upstream static pressure, P_∞ , is expressed by the following equation taken from Reference 11.

$$\frac{P_o}{P_\infty} = \left[\frac{\gamma+1}{2} M^2 \right]^{\frac{\gamma}{\gamma-1}} \left[\frac{2\gamma}{\gamma+1} M^2 - \frac{\gamma-1}{\gamma+1} \right]^{\frac{1}{1-\gamma}} \quad (D26)$$

in which γ is the specific heat ratio of air and M is the Mach number of the vehicle. When $\gamma = 1.4$, Equation D26 reduces to

$$\frac{P_o}{P_\infty} = \frac{166.92 M^7}{(7M^2 - 1)^{2.5}} \quad (D27)$$

An expression for P_∞ derived from basic definitions is as follows:

$$P_\infty = \frac{q}{\frac{1}{2} \gamma M^2} \quad (D28)$$

For air, Equation D28 reduces to

$$P_\infty = \frac{q}{0.7 M^2} \quad (D29)$$

Using the definition for pressure coefficient

$$\begin{aligned}(C_P)_{stg} &= \frac{P_o - P_\infty}{q} \\ &= \left(\frac{P_o}{P_\infty} - 1 \right) \frac{P_\infty}{q}\end{aligned}\quad (D30)$$

Combining Equations D27, D29 and D30 yields the following equation

$$(C_P)_{stg} = \frac{1}{0.7 M^2} \left[\frac{166.92 M^7}{(7M^2 - 1)^{2.5}} - 1 \right] \quad (D31)$$

The pressure coefficient, C_P , at all points on the nose cap is now defined by Equations D25 and D 31. By integrating C_P over the nose cap surface the following expression is obtained for C_D .

$$C_P = \frac{1}{2} (C_P)_{stg} (1 + \sin^2 \theta) \quad (D32)$$

in which θ is the half angle of the top frustum. The axial force contributed by the nose cap is now expressed by the following equation:

$$(\Delta F_{ax})_{CAP} = C_D q A \quad (D33)$$

in which A is the base area of the cap.

For small bluntness ratio (less than 0.2) the bending moment contribution of the nose cap can be approximated by assuming that the nose cap is replaced by a cone having the same half angle as the top frustum. C_{N_α} can then be computed by Equation C14 which is derived in Appendix C.

$$C_{N_\alpha} = 2.03 - 1.2 \theta \quad (D34)$$

Shear force contribution of the nose cap is

$$V_{CAP} = \alpha C_{N_\alpha} q A \quad (D35)$$

For a complete cone the normal force (shear force) acts a point one-third of the distance from the base of the cone to its apex.

$$\bar{X} = \frac{1}{3} \frac{d}{2 \tan \theta} \quad (D36)$$

In which d is the diameter of the nose cap base. The bending moment at the base of the nose cap is expressed as follows:

$$m_{CAP} = \bar{X} v_{CAP} \quad (D37)$$

APPENDIX E
LATERAL PRESSURE

The lateral pressure used to design the fairing is the difference between internal and external surface pressure multiplied by a factor of safety. This pressure is computed on both the windward and leeward sides of the bay, using the pressure profile data and input aerodynamic data.

Pressure coefficients on the windward and leeward sides of the bay are expressed by the following equations.

$$(C_P)_{WND} = C_{PO} + \Delta C_P \quad (E1)$$

$$(C_P)_{LEE} = C_{PO} - \Delta C_P \quad (E2)$$

In which

$(C_P)_{WND}$ = pressure coefficient on windward side

$(C_P)_{LEE}$ = pressure coefficient on leeward side

C_{PO} = pressure coefficient at zero angle of attack

ΔC_P = change in pressure coefficient due to angle of attack

The difference between surface pressure and free-stream pressure is expressed by the following equations.

$$(P_S)_{WND} - P_\infty = (C_P)_{WND} q \quad (E3)$$

$$(P_S)_{LEE} - P_\infty = (C_P)_{LEE} q \quad (E4)$$

In which

P_S = surface pressure

P_∞ = free-stream pressure

q = dynamic pressure

Recall that the difference between internal pressure and free-stream pressure is an input parameter.

$$\Delta P = P_{int} - P_\infty \quad (E5)$$

In which

ΔP = the input value of pressure difference

P_{int} = absolute pressure inside the fairing

Combining Equation E3 with E5 and E4 with E5 the following equations for pressure difference are obtained.

$$(P_S)_{WND} - P_{int} = (C_P)_{WND} q - \Delta P \quad (E6)$$

$$(P_S)_{LEE} - P_{int} = (C_P)_{LEE} q - \Delta P \quad (E7)$$

Design pressures are obtained by multiplying these pressure differences by the factor of safety, FS.

$$(P_{des})_{WND} = FS \left[(C_P)_{WND} q - \Delta P \right] \quad (E8)$$

$$(P_{des})_{LEE} = FS \left[(C_P)_{LEE} q - \Delta P \right] \quad (E9)$$

APPENDIX F
FACE WRINKLING AND DIMPLING

Face wrinkling involves the buckling of one face only. Normally, an analysis is made in a manner similar to a beam on an elastic foundation. The analysis used here (Reference 12) depends upon whether the core is "thin" or "thick". If the core is thin, it is assumed to behave like a group of independent springs with no shearing forces between them. If the core is thick, shear forces between these springs are considered. If $\frac{h}{R} > \frac{2W}{R}$ as determined by the following equation, the core is considered thick.

$$\frac{2W}{R} = \frac{2.496 t_f}{R} \left(\frac{E/G_c}{t_c/a} \right)$$

where:

- R = cylinder mean radius
- t_f = face thickness
- E = elastic modulus of face
- G_c = shear modulus of core
- t_c = core ribbon thickness
- a = free honeycomb-core wall dimension between nodes

For thick cores, wrinkling stress is predicted by

$$\sigma_\omega = 1.66 (E)^{1/3} (G_c)^{2/3} (t_c/a)^{2/3}$$

For thin cores, wrinkling stress is predicted by

$$\sigma_\omega = 1.86 (E)^{1/2} (G_c)^{1/2} (t_c/a)^{1/2} (t_f/h)^{1/2}$$

where:

- h = distance between midplanes of face sheets

The above equations predict wrinkling stress levels if there are no imperfections in the face sheets. To approximately account for the deleterious effect of reasonable imperfections, half of the predicted wrinkling stress level has been used as an allowable stress. Normally, designs having practical ribbon and face thicknesses will not be critical for wrinkling.

Face dimpling considerations limit the maximum size of the core hexagon. From Reference 12, the following equation is used to predict dimpling:

$$\sigma_d = \frac{2t_f^2 E}{a^2(1-\nu^2)}$$

where:

ν = Poisson's ratio of face sheets.

a = Minimum distance between nodes.

One half of the predicted dimpling stress value as determined by the above equation was used as an allowable stress level.

APPENDIX G
STIFFENING RING DESIGN

After the shell portion of a bay has been designed, it is necessary to provide a ring of adequate stiffness to prevent general instability of the composite structure, i.e., to prevent the entire side of the fairing from caving in. This ring is placed at the upper end of the bay. For tapered conical sections, lateral crushing pressure is normally the dominant factor in ring size determination. For these sections, the required moment of inertia of such a ring is expressed by the following equation which was used by Nevins and Helton in a similar study reported in Reference 1.

$$I_{\text{req}} = L_{\text{bay}} \left(\frac{D}{2 \cos \theta} \right)^2 \left(\frac{1}{t} \right)^{1/3} \left[\frac{(P_{\text{des}})_{\text{WND}} D_b}{11.02 E \tan \theta} \right]^{4/3} \quad (G1)$$

in which

- L_{bay} = length of bay, in.
- D = small diameter of bay, in.
- θ = semivertex angle of bay
- t = skin thickness, in.
- $(P_{\text{des}})_{\text{WND}}$ = the crushing pressure on the windward side of the fairing, psi.
- D_b = the base diameter of the fairing, in.
- E = modulus of elasticity of the material, psi.

This equation is a modification of the general stability equation developed by Becker in Reference 5.

For cylindrical sections, axial loads are higher and lateral collapse pressure much lower than in conical sections, hence rings are sized on a different basis. As stated in Section 1.13, the cylindrical section is divided into bays of equal length using stiffening rings of identical cross-section. Skin gauge for the first bay is determined by the same methods used elsewhere in the fairing. Then assuming that the skin gauge and loading determined for the first bay prevail throughout the cylindrical section, the minimum ring cross-section required to prevent general instability is computed by the method of Baruch-Singer adapted for use in this program by A. B. Burns (see Appendix K).

After computing the moment of inertia required of the ring, the ring cross-section which will provide this moment of inertia is selected. The three types of cross-sectional shapes which may be specified are shown in Figure 3. Also to be specified are B/t ratios of the web(s) and flanges. When designing the ring, the computer program selects the smallest standard skin gauge which provides a ring cross-section with moment of inertia equal to or greater than that required, providing that the selected ring has no buckled flanges.

It is necessary to check for flange (or web) buckling because large B/t values may be input, and these large B/t values present a definite possibility of local instability occurring. In making this check, a small (one percent) ovality tolerance was assumed, and bending stress due to this ovality effect are added to the hoop compression stress. The total flange stress thus obtained, is compared to an input flange buckling stress level (FCFB), and if excessive, the ring web thickness is increased as required.

APPENDIX H
STRUCTURAL DESIGN OF SPHERICAL NOSE CAP

The nose cap design is analyzed structurally as an unstiffened, non-shallow spherical cap with uniform shell thickness. The method of analysis used is presented in Section 6.23.1 of Reference 4. From experiments it is observed that non-shallow (the ratio of height to radius is greater than 1/6) spherical caps buckle in the form of a small dimple in some area of the surface of the cap. Therefore, the critical buckling pressure for non-shallow caps is independent of the height to radius ratio, depending only on the radius to thickness ratio and the modulus of elasticity of the shell material.

The equation recommended in Reference 4 is

$$P_{crt} = \frac{0.606 E}{\left(\frac{R}{t}\right)^2 e^{0.04 \sqrt{R/t}}} \quad (H1)$$

In which

P_{crt} = critical buckling pressure

R = nose cap radius

E = modulus of elasticity

t = shell thickness

A trial and error procedure is used in determining the minimum shell thickness required for the nose cap. When R , E and design pressure are known the shell thickness is increased by 0.001 inch increments until P_{crt} is equal to or greater than the design pressure computed for the nose cap.

Maximum design pressure for the nose cap occurs at the stagnation point. Assuming that the pressure on this point is the same as the stagnation pressure downstream from a normal shock, the pressure coefficient, $(C_p)_{stg}$, is expressed by Equation D30, and the design pressure is expressed by the following equation:

$$(P_d)_{CAP} = (FS) \left[q (C_p)_{stg} - \Delta P \right] \quad (H2)$$

in which

FS = factor of safety

q = dynamic pressure

ΔP = internal to free-stream pressure difference

APPENDIX I
EFFECTIVE SHEAR MODULUS OF
HONEYCOMB CELLULAR STRUCTURE

The general instability analysis developed for this program accounts for the effect of transverse shear deformation, hence buckling loads are a function of core shear stiffness. Core shear stiffness is calculated for aluminum hexagon honeycomb using the analysis of Penzien and Didriksson (Reference 13). The effective shear modulus is

$$G_c = \frac{G \sin \beta (R + \cos \beta)}{\frac{A}{t_r} \left[(1 + R) \sin^2 \beta \cos^2 \phi + (R + \cos \beta)^2 \sin^2 \phi \right]}$$

where:

G = shear modulus of core material

$R = B/A$ (See Figure 1 below)

t_r = thickness of core material

ϕ = angle between direction of the shear force and ribbon direction

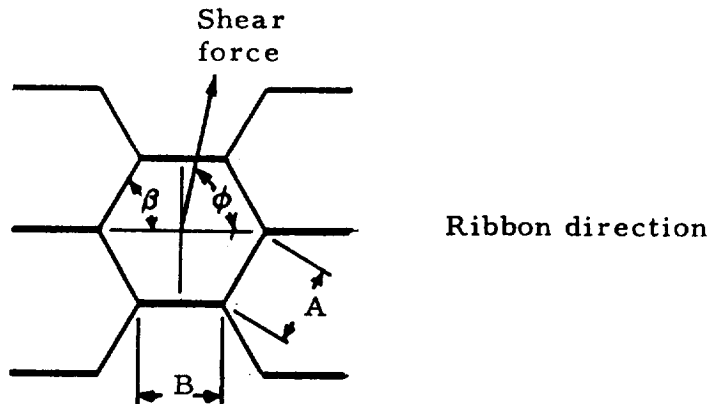


Figure 1 - Geometry of honeycomb cells

A correction factor based on Figure 6 of Reference 13 is also applied to the above equation when the ratio of core height (H) to average cell wall length ($\frac{1}{2}A + \frac{1}{2}B$) is 5 or less.

When the ratio of core height to the quantity $W = 0.5 (A + B)$ is five or less, prevention of warpage increases the effective shear modulus (See Figure 6 of Reference 13). For values of H/W less than five, the effective shear modulus is calculated from the equation

$$G_{\text{ceff}} = G_c (1 + 0.01Y)$$

where:

$$Y = (9.3/(H/W)) - (2.1/(H/W)^2)$$

APPENDIX J
THERMAL ANALYSIS

The maximum temperature reached by the skin due to aerodynamic heating can be controlled by placing a lower limit on the thickness of skin to be used in critical locations. Provisions have been made in the program to place such constraints on both the nose cap and top frustum of the fairing, either by specifying the minimum skin thickness to be used or by specifying the maximum temperature to be reached by the skin.

Design curves for determining the minimum thicknesses have been prepared by LMSC/HREC. These curves and the methods used in generating the data for these curves are presented in Reference 6. The trajectory used in this analysis was a nominal two-stage Saturn V ascent trajectory to a 100 nautical mile circular orbit.

Both laminar and turbulent heating occur during the flight. Laminar flow was assumed to exist when the Reynolds number based on momentum thickness of the local boundary layer was equal to or less than 500. Turbulent flow was assumed to exist at Reynolds numbers above 500. When the flow was laminar, the method of Fay and Riddell (Reference 7) was used together with the laminar heating rate distribution of Lees (Reference 8). When the flow was turbulent, heating rates were calculated by using a method from Reference 9 (Bromberg, Fox and Ackermann). Radiation from the outer surface was also taken into account.

Other assumptions were that the heat flow is one-dimensional, that at any time or location on the fairing the skin temperature is uniform throughout the thickness of the skin, and that the inner surface of the skin is perfectly insulated. These latter assumptions were found to have only a minor effect on the final results.

Maximum temperature constraints are applied to the nose cap and the top frustum. The thickness of material required to limit the maximum temperature of the nose cap is based on heating at the stagnation point of the nose cap, and the thickness required for the top frustum is based on heating on the nose cap at its junction with the top frustum. Thus, the heating data required to establish these constraints can be obtained from a spherical shell.

Several hundred data points were generated for each of the following five materials: aluminum, magnesium, titanium, stainless steel and Lockalloy. Each data point for a specified material is completely described by the following four parameters:

- R = radius of spherical nose cap
- θ = the angle between the line of flight and a plane tangent to the nose cap at the point of interest

$T_{\max}$ = the maximum temperature reached by the skin, $^{\circ}\text{F}$

t = thickness of the skin

When applying this data to nose fairing design, θ is equal to 90° at the stagnation point and to the half angle of the top frustum at the junction of the nose cap and top frustum. (Note that ϕ in Reference 6 is the complement of θ .)

In order to avoid the necessity of storing all of these data points in the fairing design program, a set of linear algebraic equations which fit the data within a few percent was developed by a technique commonly referred to as multiple regression analysis. A detailed description of multiple regression analysis can be found in many statistical text books such as Reference 10, Chapters 4 and 5.

Two major steps are involved in such an analysis. First, it is necessary to establish the form of the equation relating the variables which describe the data points. This step can be based on intuition and/or a knowledge of the physical laws relating the variables. This equation must be reduced to linear form, which is then referred to as a linear model. The linear model has the general form

$$y = c_1 x_1 + c_2 x_2 + \dots + c_n x_n \quad (J1)$$

The variables y and $x_1, x_2, \dots, x_n$ may be grouped parameters such as $(T_{\max} - 70)$ and $T_{\max} / \sqrt{R}$. However, these variables must be such that numerical values can be obtained for each variable for each data point.

Having developed a linear model the next step is to determine the set of coefficients ($c_1, c_2, \dots, c_n$) which give the best fit to the data. This is done by means of the "least squares" curve fit technique. Usually several different linear models are tried in an attempt to curve-fit a given set of data.

For this application two models were developed, one for the stagnation point and the other for the point of tangency between the spherical nose cap and the top frustum. A set of coefficients was computed for each of the two models for each of the five materials, making a total of ten sets of coefficients. The equations and coefficients appear in Subroutine THERML.

The linear model representing heating at the stagnation point is based on laminar and radiative heating theory. It is postulated that the following

relationship is approximately true.

$$\rho C_P t (T_{\max} - 70) = K_1 + \frac{K_2}{\sqrt{R}} (K_3 - T_{\max}) - K_4 \epsilon (T_{\max} + 460)^4 \quad (J2)$$

in which

ρ = density of skin

C_P = specific heat of skin

ϵ = emissivity of skin

The terms in Equation J2 represent the following physical quantities:

$C_P t (T_{\max} - 70)$ = the maximum quantity of heat stored in a unit area of skin during the flight

K_1 = a constant

$\frac{K_2}{\sqrt{R}} (K_3 - T_{\max})$ = convective heat input (laminar flow) to the unit area

$K_4 \epsilon (T_{\max} + 460)^4$ = radiative heat loss from the unit area

When material properties are dropped (coefficients are determined for each material) and when the multiplications are performed Equation J2 reduces to the following linear form.

$$t (T_{\max} - 70) = a_1 + a_2 \frac{1}{\sqrt{R}} + a_3 \frac{T_{\max}}{\sqrt{R}} + a_4 (T_{\max} + 460)^4 \quad (J3)$$

Comparing this to Equation J1

$y = t (T_{\max} - 70)$

$x_1 = 1$

$$x_2 = \frac{1}{\sqrt{R}}$$

$$x_3 = \frac{T_{\max}}{\sqrt{R}}$$

$$x_4 = (T_{\max} + 460)^4$$

Using the data generated in Reference 6 the coefficients a_1 , a_2 , a_3 and a_4 can be determined by the least-squares technique.

A more complex model is required to represent heating at the point at which the nose cap is tangent to the top frustum. Turbulent flow occurs during part of the flight, and laminar flow occurs during the remainder. It is also necessary to specify the angular location, θ , on the sphere. The following model was postulated:

$$\begin{aligned} \rho C_P t (T_{\max} - 70) = & K_1 + \frac{K_2}{\sqrt{R}} (K_4 + \sin^2 \theta) (K_5 - T_{\max}) \\ & + \frac{K_3}{R^{0.2}} (K_4 + \sin^2 \theta) (K_5 - T_{\max}) - K_6 \epsilon (T_{\max} + 460)^4 \end{aligned}$$

The additional terms in this equation have the following physical significance:

$$\frac{K_2}{\sqrt{R}} (K_4 + \sin^2 \theta) (K_5 - T_{\max}) = \text{convective heat input during laminar flow}$$

$$\frac{K_3}{R^{0.2}} (K_4 + \sin^2 \theta) (K_5 - T_{\max}) = \text{convective heat input during turbulent flow}$$

in which $(K_4 + \sin^2 \theta)$ accounts for pressure variation with angular position on the nose cap. When material properties are dropped and the multiplications are performed the quantities corresponding to the variables in the linear model are as follows:

$$y = t (T_{\max} - 70)$$

$$x_1 = 1$$

$$x_2 = \frac{1}{\sqrt{R}}$$

$$x_3 = \frac{T_{\max}}{\sqrt{R}}$$

$$x_4 = (T_{\max} + 460)^4$$

$$x_5 = \frac{\sin^2 \theta}{\sqrt{R}}$$

$$x_6 = \frac{T_{\max} \sin^2 \theta}{\sqrt{R}}$$

$$x_7 = \frac{1}{R^{0.2}}$$

$$x_8 = \frac{T_{\max}}{R^{0.2}}$$

$$x_9 = \frac{\sin^2 \theta}{R^{0.2}}$$

$$x_{10} = \frac{T_{\max} \sin^2 \theta}{R^{0.2}}$$

Note that y and x_1 , x_2 , x_3 , and x_4 are identical for both the stagnation point and tangency point models.

Coefficients were determined for each of the five materials at both the stagnation point and the tangency point. A summary of pertinent information about the curve fit is presented in Table J1.

The coefficients stored in Subroutine THERML were determined for a nominal two-stage Saturn V ascent trajectory to a 100 nautical mile orbit. If the subroutine is to be used for trajectories which differ greatly from this trajectory it would be advisable to determine a new set of coefficients based on thermal data for the new trajectory. The linear models used in Subroutine THERML will probably be valid for a wide range of trajectories.

TABLE JI
THERMAL CURVE-FIT INFORMATION

Location	Material	Range of Data				No. of Data Points	Range of Errors in Skin Thickness, Percent	
		Temperature, °F		Skin Thickness, in			Negative	Positive
		Low	High	Low	High			
Stagnation Point	Aluminum	366	987	0.050	0.400	28	-0.66	0.50
	Magnesium	388	1000	0.060	0.500	28	-0.93	0.42
	Titanium	579	1234	0.040	0.250	21	-0.39	0.30
	Stainless Steel	746	1361	0.025	0.080	23	-0.91	1.26
	Lockalloy	369	1175	0.035	0.300	30	-1.50	0.78
Tangency Point	Aluminum	404	1091	0.025	0.200	124	-3.44	3.84
	Magnesium	396	1132	0.020	0.300	149	-7.00	4.81
	Titanium	475	1365	0.015	0.200	125	-3.49	3.79
	Stainless Steel	647	1147	0.0125	0.060	72	-4.55	7.84
	Lockalloy	395	1234	0.025	0.150	105	-6.03	7.20

NOTES:

Nose cap radius ranges from 13 to 52 inches for both locations and all materials.

Angular location of the tangency point (half angle of the top frustum) ranges from 0 to 45 degrees for all materials for the tangency point data.

APPENDIX K
GENERAL INSTABILITY ANALYSIS OF
HONEYCOMB SANDWICH CYLINDERS

by

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GENERAL INSTABILITY ANALYSIS OF HONEYCOMB SANDWICH CYLINDERS

Summary

In the establishment of the general instability load for a sandwich shell, the practical method of analysis which was recommended in Ref. 1 is used. This method uses the classical buckling load as an upper bound and the minimum postbuckling load as a lower bound to the critical load of the shell. An empirical reduction factor is applied to the upper bound in such a manner that the critical load will not fall below the lower bound. For the case of pure axial compression upper as well as lower bound analyses are available in Ref. 2. The classical buckling load analysis is modified here to include the effects of lateral pressure but the reduction factor has been chosen in a more expedient manner.

Classical Buckling Load

For sandwich cylinders under pure axial compression it was found in Ref. 3 that buckling can occur either in an axisymmetrical or in a nonsymmetrical mode, the former being critical for cylinders with weaker cores. It may be shown that the critical axial load corresponding to symmetrical buckling is independent of the lateral pressure. On the other hand the nonsymmetrical buckling load is reduced in the presence of an external pressure. According to Ref. 2, for the symmetrical pattern:

$$\bar{N} = (1 + F)z + 1/(4z) - 4z^2/[4z + (1/G)]$$

Here $\bar{N}$ is to be minimized with respect to the wavelength parameter z .

After addition of the influence of external pressure, we have for the non-symmetrical buckling mode:

$$\bar{N} = (1 + F)(1 + \beta^2)^2/(4\beta^2/\eta) + (\beta^2/\eta)/(1 + \beta^2)^2 \\ + (b_2^2 + \mu c_2^2)/(4\beta^2/\bar{G}) + s_2/(8\beta^2/\eta) - \bar{p}/\beta^2$$

where

$$s_2 = 2[\beta^2 b_1^2 + \mu^2 c_2^2 + 2\nu\mu\beta b_2 c_2 + 2(1 - \nu)(b_2 + \mu\beta c_2)^2 \\ + 4[b_2\beta(2 - \nu + \beta^2) + c_2\mu\{1 + (2 - \nu)\beta^2\}]$$

$$b_1 = -8\beta^3/[1/(\bar{G}\eta) + 4\beta^2] \quad (1)$$

$$b_2 = -\beta(1 + \beta^2)\{1/(\bar{G}\eta) + (1 - \nu)\mu\beta^2\}/[\{1/(\bar{G}\eta) + (1 - \nu)\mu\beta^2 + \mu\} \\ \{1/(\bar{G}\eta) + 1 - \nu + \beta^2\} - \mu\beta^2]$$

$$c_2 = -(1 + \beta^2)\{1/(\bar{G}\eta) + 1 - \nu\}/[\{1/(\bar{G}\eta) + (1 - \nu)\mu\beta^2 + \mu\} \\ \{1/(\bar{G}\eta) + 1 - \nu + \beta^2\} - \mu\beta^2]$$

The nonsymmetrical buckling load is found through minimization of $\bar{N}$ with respect to the wavelength parameters η and β . These parameters can only take on the specific values which correspond to an integer value for the number of waves in the circumferential and the number of halfwaves in the axial direction. However, we will only apply here the restriction that the axial halfwave length cannot be larger than the shell length. That is

$$\eta \leq \alpha/\beta^2$$

where, for equal face sheets

$$\alpha = \pi^2 \sqrt{1 - \nu^2} Rh/L^2$$

In the computer program the symmetrical buckling load is first determined. The equation $\partial \bar{N} / \partial z = 0$ is solved by use of the Newton-Raphson method. As initial estimates for z are used

$$z = \begin{cases} 1/(2 - 4\bar{G}) & \text{if } \bar{G} < .5 \\ 1/\sqrt{4F} & \text{if } \bar{G} \geq .5 \end{cases}$$

It appears that the Newton-Raphson method would be most efficient whenever reasonably close estimates are available. In the procedure of optimization close estimates are generally available in the form of previous solutions. In such cases the Newton-Raphson method is used directly. However, when a new case is started close initial estimates are not available and the procedure is modified as follows. The initial estimates are

$$\beta = 1$$

$$\eta = \begin{cases} 2\alpha & \text{if } \alpha \geq 1 \\ 1 & \text{if } \alpha < 1 \end{cases}$$

These estimates are improved by use of the steepest descent method before they are used in the Newton-Raphson method. If during iteration η becomes less than α/β^2 this indicates that buckling will be with one halfwave in the axial direction. Consequently we set $\eta = \alpha/\beta^2$ and minimize with respect to β only. After minimization the proper value of $\bar{N}$ is computed and the classical buckling load is chosen as the lowest of the symmetrical and non-symmetrical buckling loads.

Reduction Factor

Inclusion of lateral pressure in the lower bound analysis would be a major undertaking and was not considered necessary for the purpose of this study. For monocoque cylinders it was shown in Ref. 3 that the ratio between

lower and upper bound increases when an internal pressure is added. It seems reasonable to assume that this is the case also for sandwich cylinders. It is well known also that for cylinders under external pressure the agreement between theory and test is much better than it is for cylinders under axial compression. Consequently it seems safe to assume that a conservative estimate of the reduction factor will be obtained if the effect of lateral pressure is neglected. It is expected that the shells in this investigation will be primarily subjected to axial compression and thus the conservatism involved will be moderate.

When the effects of lateral pressure are neglected, the ratio $N_{\min}/N_{cl}$ may be obtained from Ref. 2. A number of values of $N_{\min}/N_{cl}$ are read from the curves of that analysis and stored in the computer. These values correspond to different shell parameter combinations in such a manner that intermediate results can be obtained easily through interpolation.

According to Ref. 1:

$$N_x = N_{cl} \left[\frac{N_{\min}}{N_{cl}} + c \left(1 - \frac{N_{\min}}{N_{cl}} \right) \right]$$

where

$$c = \frac{\varphi - 0.12}{0.88} \quad (2)$$

$$\varphi = \begin{cases} 1.0 & \text{for } (R/t)_e \leq 33 \\ 6.48/[(R/t)_e] & \text{for } (R/t)_e > 33 \end{cases}$$

For equal face sheets

$$(R/t)_e = R/\sqrt{t^2 + 3h^2}$$

Nomenclature

R	mean radius of cylinder
E	Young's modulus of face sheets
ν	Poisson's ratio of face sheets
t	thickness of face sheets
h	distance between midplanes of face sheets
G_1	shear modulus of core in axial direction
G_2	shear modulus of core in circumferential direction
ℓ_x, ℓ_y	axial and circumferential half-wave lengths
p	external pressure
N	axial compressive load per unit width
$\bar{N}$	$RN(1 - \nu^2)^{1/2}/(2Eht)$
$\bar{p}$	$R^2p(1 - \nu^2)^{1/2}/(Eht)$
F	$(t/h)^2/3$
$\bar{G}$	$1/2(E/G_1)(t/R)(1 - t/h)(1 - \nu^2)^{-1/2}$
μ	G_1/G_2
α	$\pi^2(1 - \nu^2)^{1/2}(Rh/L^2)$
z	θ^2/η
θ	ℓ_y/ℓ_x
η	$\pi^2(1 - \nu^2)^{1/2}(Rh/\ell_y^2)$
$N_{\min}/N_{cl}$	ratio between lower and upper bounds for critical load
c	see Eq. 2
φ	see Eq. 2
$b_1, b_2,$ c_2, s_2	see Eq. 1
$(R/t)_e$	$R/\sqrt{t^2 + 3h^2}$

Subroutine Symbols

CRG1	critical buckling line load (pounds/in)
E	Young's modulus of facing sheets
G1	Effective shear modulus of core in axial direction
G2	Effective shear modulus of core in circumferential direction
HT	distance between midplanes of sandwich facing sheets
K	optional control; used to speed up calculations for classical buckling load if desired
P	collapse pressure (psi); burst pressure is entered as negative quantity
RD	cylinder mean radius
T1, T2	thicknesses of face sheets
XL	cylinder length
XNU	Poisson's ratio of facings material

References

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