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**SAMMSOR II - A FINITE ELEMENT
PROGRAM TO DETERMINE STIFFNESS AND
MASS MATRICES OF SHELLS OF REVOLUTION**



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ABSTRACT

A user's guide for the SAMMSOR II (Stiffness And Mass Matrices for Shells Of Revolution) computer code is presented in this report. The finite element method of analysis is employed using a curved shell of revolution element to determine structural stiffness and mass matrices. Documentation of the analysis technique is included along with user hints and a discussion of the program limitations. Instructions for preparing the input data is included along with procedures for modifying the code. This program has the capability of internally generating the geometry of a number of important classes of shell configurations (such as shallow caps, cylinders, cones, hemispheres, etc.). Several example problems are presented to aid the user.

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SOR - Shell Of Revolution

Computer Programs

A family of compatible computer codes for the analysis of the shell of revolution (SOR) structures has been developed by researchers at Texas A&M University. These analyses employ the matrix displacement method of structural analysis utilizing a curved shell element. Geometrically nonlinear static and dynamic analyses can be conducted using these codes. The important natural frequencies and mode shapes can also be determined by employing another of the codes. Efficient programming provides codes capable of performing these desired analyses in relatively small amounts of computer time.

Each of these programs has been extensively tested using problems the solutions to which have been reported by other researchers in order to establish the validity of the codes. In addition, the capabilities of the codes have been demonstrated in a number of publications by presenting solutions to problems which were unsolved by other researchers.

SAMMSOR II - Stiffness And Mass Matrices for Shells Of Revolution are generated utilizing the first member of this family. This program accepts a description of the structure in terms of the coordinates and slopes of the nodes and the properties of the elements joining the nodes. For shells with simple geometries (such as cylinders, shallow caps, hemispheres, etc.) the shell geometry can be internally generated. Utilizing the element properties, the structural stiffness and mass matrices are generated for as many as twenty harmonics and stored on magnetic tape. Each of the other SOR programs utilizes the output tape generated by SAMMSOR as input

data for the respective analyses. One advantage of creating the stiffness and mass matrices in a separate program is that a variety of analyses can be performed on the same shell configuration without having to create the matrices more than once. Obviously, a variety of boundary and loading conditions can be employed without having to create new mass and stiffness matrices for each case.

SNASOR II - The Static Nonlinear Analysis of Shells Of Revolution subjected to arbitrary mechanical and thermal loading is performed using the second computer code. Utilizing the stiffness matrices generated by SAMMSOR and the loading conditions and boundary conditions input to SNASOR II, the equilibrium equations for the structure are generated. The nonlinear strain energy terms result in pseudo generalized forces (as functions of the displacements) which are combined with the applied generalized forces. The resulting set of nonlinear algebraic equilibrium equations is solved by one of several methods: Newton-Raphson type iteration, incremental stiffness method, or a modified incremental stiffness method. In general, the Newton-Raphson procedure is the best and yields accurate results for highly nonlinear problems.

DYNASOR II - The third code is used for the DYnamic Nonlinear Analysis of Shells Of Revolution. The equations of motion of the shell are solved using Houbolt's numerical procedure with the nonlinear terms being moved to the right-hand side of the equilibrium equations and again treated as generalized loads. The displacements and stress resultants can be determined for both symmetrical and asymmetrical loading conditions. Asymmetrical dynamic buckling can be investigated using this program. Solutions can be obtained for highly nonlinear problems in reasonable periods of time on the computer utilizing as many as five of the harmonics generated

in SAMMSOR. A restart capability is incorporated in this code which allows the user to restart the program at a specified time without having to expend the computer time necessary to regenerate the prior response.

FAMSOR - Frequencies And Modes for Shells Of Revolution can be determined using the fourth code. Using the stiffness matrix generated by SAMMSOR and a lumped mass representation developed from the consistent mass matrix generated by SAMMSOR, a specified number of natural frequencies (beginning with the lowest or fundamental frequency) are obtained using the inverse iteration method. The mode shapes for each of the frequencies are also obtained.

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NOMENCLATURE

Variable

a_1, a_2, a_3 = coefficients in the expansion for shell slope representation

$$C_1 = E_s t / (1 - \nu_{s\theta} \nu_{\theta s})$$

$$C_2 = E_\theta t / (1 - \nu_{s\theta} \nu_{\theta s})$$

$$D_1 = E_s t^3 / [12(1 - \nu_{s\theta} \nu_{\theta s})]$$

$$D_2 = E_\theta t^3 / [12(1 - \nu_{s\theta} \nu_{\theta s})]$$

E = modulus of elasticity

e = linear strains and rotations of shell middle surface

G = shear modulus

$$G_1 = Gt$$

$$G_2 = Gt^3/12$$

IA = number of Fourier cosine harmonics

$[K]$ = structural stiffness matrix

$[M]$ = structural mass matrix

q = generalized nodal displacement (cylindrical coordinates)

r = radial coordinate normal to the axis of revolution

s = meridional coordinate

T = kinetic energy

U = strain energy

u, v, w = meridional, tangential, and normal displacements, respectively

z = axial coordinate

α = generalized coefficient of a displacement function

θ = circumferential angular coordinate

ν = Poisson's ratio

ϕ = angle between meridian and axis of revolution in the undeformed shell

χ = changes in curvature

σ = stress component

INTRODUCTION

The SAMMSOR (Stiffness And Mass Matrices for Shells Of Revolution) II code has been developed as the basic code used by the three SOR family analysis programs --- SNASOR II, DYNASOR II, and FAMSOR. This report is a user's guide for the SAMMSOR II code and is divided into four self-contained sections with an extended appendix.

The first section of the report describes the method of analysis used to obtain the output matrices. This documentation is not intended to be a detailed derivation of the element and structural properties but is intended to show the basic sets of equations employed in calculating the element and structural properties.

A section providing guidelines for the user and enumerating many of the limitations of the code is then presented. The hints contained in this second section concern mainly the selection of an element idealization for the shell. The limitations presented result partly from the method of analysis and partly from the programming procedures and storage capacity utilized.

A description of the input data required by the code is presented in the third section. Limitations placed upon the input parameters are once again presented and examples are provided in cases where the wording might, at first glance, seem to be unclear or insufficient.

A final section containing a number of example problems follows the description of the input data. Copies of the input data required for each case are presented along with selected output that may be scrutinized by a user who desires to check his output. The user should study and understand the example problems before attempting to input to the code.

While the appendix which follows the main report may seem to be quite extensive, it is believed that the user will find the various sections extremely useful, especially if a thorough understanding of the program is desired. A description of the various subroutines, a glossary defining the significant FORTRAN variables, a subroutine call map and a flow chart of the basic operations are included in this appendix. A discussion of the program output is also included. Since undoubtedly some users will find it necessary to modify the element capacity of the code, a section in the appendix describes this procedure. A sample coding sheet for use in preparing input data is provided to aid the user.

An extended effort has been made to anticipate the questions which a user might ask before running the program. In particular, the report attempts to provide guidelines and hints for the successful use of the options for inputting the shell geometry and for the selection of the element idealization.

SECTION I

METHOD OF ANALYSIS

Introduction

The purpose of this section is to provide theoretical documentation of the equations and procedures employed in the SAMMSOR II code to calculate the mass and stiffness matrices used by the Shell Of Revolution (SOR) analysis programs. This program uses the curved shell element of Stricklin, Navaratna, and Pian¹ and the displacement function investigated by Mebane.² Since the nonlinear terms are treated as pseudo generalized forces in both the static³ and dynamic⁴ nonlinear analyses, the stiffness and mass matrices remain the same as in the linear analysis of shells of revolution. Detailed presentations of the methods of linear analysis can be found in Refs. 1, 2, and 5.

Structural Idealization

The shell of revolution is idealized as a sequence of consecutively numbered curved elements. The slope of each element (Fig. 1) is represented by a second-order polynomial in the meridional distance s :

$$\phi = a_1 + a_2 s + a_3 s^2 \quad (1)$$

where ϕ is the slope between the vertical axis and a tangent to the shell in the meridional direction, s is the meridional distance along the element, and a_1 , a_2 , and a_3 are coefficients that are evaluated by requiring the slopes of the structural idealization and the actual shell to be the same at the nodal points.

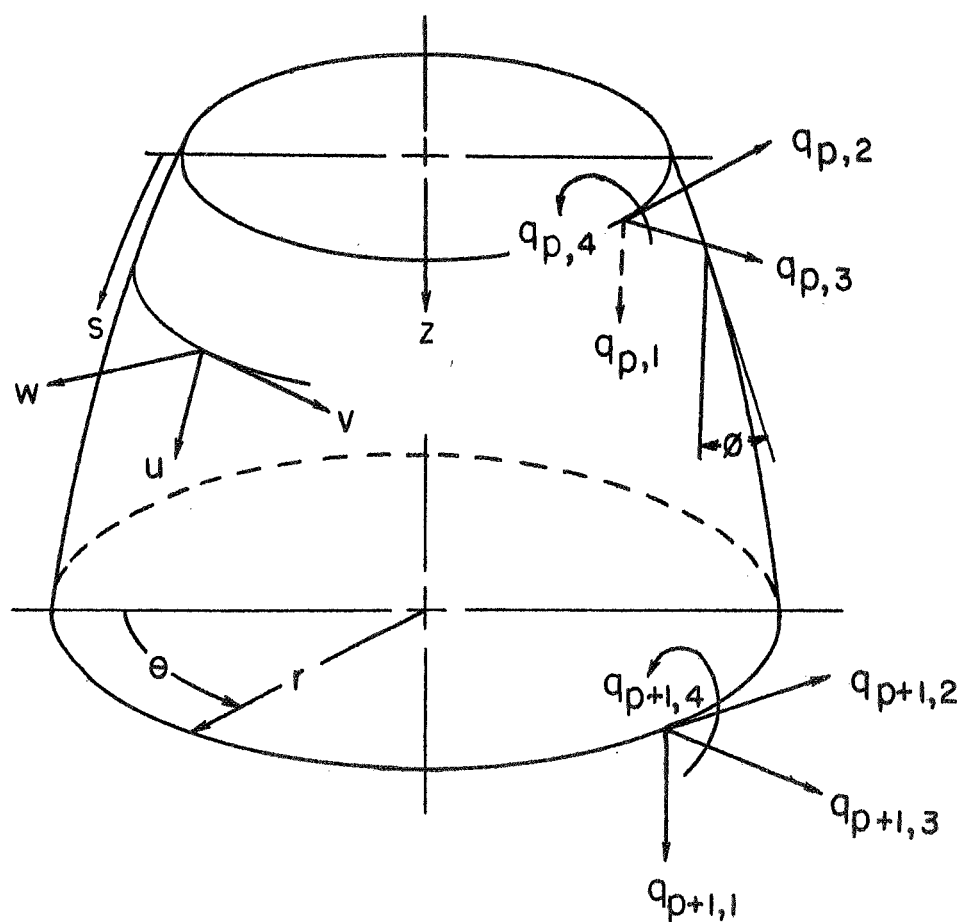


Fig. 1 GENERALIZED COORDINATES
OF SHELL ELEMENT

Displacement Functions

The displacements of an element in the normal, meridional, and circumferential directions are represented by cubic polynomials in the meridional distance s and by a Fourier expansion in the circumferential angle θ :

$$\begin{aligned}
 w &= \sum_{i=0}^{IA} [\alpha_1^i + \alpha_2^i s + \alpha_3^i s^2 + \alpha_4^i s^3] \cos i\theta \\
 u &= \sum_{i=0}^{IA} [\alpha_5^i + \alpha_6^i s + \alpha_9^i s(s-l) + \alpha_{10}^i s^2(s-l)] \cos i\theta \\
 v &= \sum_{i=0}^{IA} [\alpha_7^i + \alpha_8^i s + \alpha_{11}^i s(s-l) + \alpha_{12}^i s^2(s-l)] \sin i\theta
 \end{aligned} \tag{2}$$

In these expressions, the α 's are the generalized coefficients; l is the length of an element in the meridional direction; and i is the harmonic number.

Four degrees of freedom of the element are eliminated using the technique of static condensation.⁶ After condensation each element has an 8×8 element stiffness matrix. The net result is therefore a stiffness matrix based upon higher order displacement functions without requiring additional geometrical data about the elements.

Linear expressions for u and v are used in the SAMMSOR I code,⁴ but it has been recently shown² that the cubic polynomials better represent rigid-body motion and converge somewhat faster than the linear displacement fields. The input data for the SAMMSOR I and II codes is identical.

In both the SNASOR II and DYNASOR II codes, linear displacement functions are used for evaluating the nonlinear terms; justification for this is provided in Ref. 4.

The generalized coordinates, q_i , of the shell are defined as the displacements and rotations at the nodes in cylindrical coordinates for each harmonic (Fig. 1). The shell generalized coordinates are related to the generalized coefficients, α , through the relation

$$\{\alpha\} = [A]\{q\} \quad (3)$$

where

$$[A] = [B]^{-1}[\Psi]'$$

The matrices $[B]^{-1}$ and $[\Psi]'$ are presented as expressions (5) and (6) in Ref. 3.

Strain-Displacement Relations

While the nonlinear shell theory of Novozhilov⁷ is applied in the SNASOR and DYNASOR codes the method of treating the nonlinear terms allows calculation of the stiffness matrix for each harmonic based upon the following linear strain-displacement relations for the midsurface strains:

$$\begin{aligned} e_s &= (\partial u / \partial s) - \phi' w \\ e_\theta &= (1/r)[(\partial v / \partial \theta) + u \sin \phi + w \cos \phi] \\ e_{s\theta} &= (1/r)(\partial u / \partial \theta) - (v/r) \sin \phi + \partial v / \partial s \end{aligned} \quad (4)$$

where

$$\phi' = \frac{\partial \phi}{\partial s}$$

The changes in curvature of the shell element are given by:

$$\begin{aligned} \chi_s &= -\partial \hat{e}_{13} / \partial s \\ \chi_\theta &= -(1/r)(\partial \hat{e}_{23} / \partial \theta) - (1/r) \sin \phi \hat{e}_{13} \\ \chi_{s\theta} &= -(1/r)(\partial \hat{e}_{13} / \partial \theta) + (1/r) \sin \phi \hat{e}_{23} - \partial \hat{e}_{23} / \partial s \end{aligned} \quad (5)$$

where

$$\begin{aligned}\hat{e}_{13} &= (\partial w / \partial s) + u \phi' \\ \hat{e}_{23} &= (1/r)(\partial w / \partial \theta) - (v \cos \phi)/r\end{aligned}$$

The strains at any point through the thickness of the shell can be written as

$$\begin{aligned}\epsilon_s &= e_s + \chi_s z \\ \epsilon_\theta &= e_\theta + \chi_\theta z \\ \epsilon_{s\theta} &= e_{s\theta} + \chi_{s\theta} z\end{aligned}\tag{6}$$

Stress-Strain Relations

The stress-strain relationships used in this analysis are valid for an orthotropic material whose principal lines of orthotropy are the s and θ directions. These expressions can be written as:

$$\begin{aligned}\sigma_s &= \frac{E_s}{1 - \nu_{s\theta}\nu_{\theta s}} (\epsilon_s + \nu_{s\theta}\epsilon_\theta) \\ \sigma_\theta &= \frac{E_\theta}{1 - \nu_{s\theta}\nu_{\theta s}} (\epsilon_\theta + \nu_{\theta s}\epsilon_s) \\ \sigma_{s\theta} &= G \epsilon_{s\theta}\end{aligned}\tag{7}$$

Strain Energy

The strain energy expression for orthotropic shells is given by

$$U = \frac{1}{2} \iiint [\sigma_s \epsilon_s + \sigma_\theta \epsilon_\theta + \sigma_{s\theta} \epsilon_{s\theta}] r d\theta ds dz\tag{8}$$

Substituting Eqs. 7 into Eqs. 8 and integrating through the thickness,

the strain energy can be expressed in the following form

$$U = \frac{1}{2} \iint (C_1 \epsilon_s^2 + C_2 \epsilon_\theta^2 + 2\nu_{s\theta} C_1 \epsilon_s \epsilon_\theta + G_1 \epsilon_{s\theta}^2 + D_1 \chi_s^2 + D_2 \chi_\theta^2 + 2\nu_{s\theta} D_1 \chi_s \chi_\theta + G_2 \chi_{s\theta}^2) r d\theta ds \quad (9)$$

The strain energy can also be written as a quadratic form and the element stiffness matrix calculated using

$$U^i = [q^i] [k^i] \{q^i\} \quad (10)$$

where i denotes the harmonic number. This procedure has been explained in detail in Refs. 1 and 3.

Mass Matrix

The element mass matrices, which include the effects of rotary inertia, can be obtained by considering the expression for the kinetic energy:

$$T = \frac{1}{2} \int_{V_0} (\dot{u}^2 + \dot{v}^2 + \dot{w}^2) dm \quad (11)$$

Substituting the proper derivatives of the displacement functions, the kinetic energy can be written as a quadratic form in the generalized velocity coefficients.

$$T = \frac{1}{2} [\dot{\alpha}] [\bar{M}] \{\dot{\alpha}\} \quad (12)$$

The $\{\dot{\alpha}\}$ matrix is related to the velocities of the generalized structural coordinates, $\{\dot{q}\}$, by the transformation

$$\{\dot{\alpha}\} = [A] \{\dot{q}\} \quad (13)$$

The terms in $[\bar{M}]$ are listed in the appendix of Ref. 4. Because of the inclusion of the effects of rotary inertia, the mass matrix is a function of

the harmonics being used. The mass matrix in structural coordinates is then obtained from

$$[M^e] = [A]^T [\bar{M}] [A] \quad (14)$$

The mass matrices for the individual elements are then combined (in the same way as for element stiffness matrices) to obtain the structural mass matrix.

SECTION II

Guidelines and Program Limitations

Guidelines for the use of the SAMMSOR II code along with the limitations placed on the analysis are enumerated in this section. Some of these limitations result from the method of analysis utilized while others result from the programming procedures which are used. Most of these limitations are minor in nature and hence this code, when coupled with the other compatible SOR programs, can be used to analyze a large number of practical shell of revolution problems.

First of all, the shell must be idealized as a sequence of curved elements. As is the case with most computer codes using the finite element method, the selection of the element breakdown of the structure requires the application of engineering judgement. The following considerations should prove most helpful in the selection of the idealization:

- (1) Elements should be closely spaced in regions where rapid variations of the displacements and stresses are anticipated.
- (2) Elements should be concentrated in regions of rapidly varying material properties.
- (3) The change in slope over an element should, in general, be kept less than about 10° .

Examples of judicious choices of element breakdown can be seen by considering the example problems presented in Section IV. If doubt still exists about how many elements to use, it is recommended that the maximum number of elements allowed by the program (50) be employed.

The material used in the shell may be isotropic or orthotropic. For an orthotropic material, the principal directions of orthotropy must be in

the meridional and circumferential directions. It is also assumed that the material properties for any given element are constant in the two directions, but these properties may vary from element to element. The properties of an element must be constant through the thickness of the element. The thickness of an individual element is constant, but thickness variations from element to element are allowed.

Slope discontinuities are not allowed between elements, i.e. the slope at the end of element i must equal the initial slope of element $i + 1$.

The code requires that the units for the input, and hence output, data be given in inch-pound-seconds. A different set of consistent units may be utilized if the value of the FORTRAN variable GRAVITY is changed in subroutines AZERO and NONAB.

The integrals which are required for the calculation of the element stiffness and mass matrices are evaluated exactly in the circumferential direction, but Simpson's rule is applied at twenty-nine (29) equally spaced stations along the element in the meridional direction. Fewer stations could probably be used for most problems, but good results have been obtained in all cases using 29 stations.

Stiffness and mass matrices may be generated for the first 20 Fourier harmonics. In general, it has been found that few problems require more than 3-5 of the harmonics. Unless the exact number of harmonics to be used by the solution programs is known, it is better to generate a relatively large number of harmonics say (6-8) in SAMMSOR so that they will be available if needed.

Utilizing only the cosine harmonics necessitates that the meridional line traced by $\theta = 0$ be selected so that the displacements, u and w , of the shell will be symmetrical about this line in the θ - direction.

A maximum of fifty (50) elements are allowed in the SAMMSOR code. It is believed that 50 elements is sufficient for a very large number of shell of revolution problems. More elements may be needed for analyzing geometrically complex structures such as the shell with negative Gaussian curvature depicted in Fig. 2-d of Ref. 3. Since undoubtedly some users will find it necessary to change the maximum allowable number of elements in the program, instructions for changing the capacity are given in Appendix 6.

A different output tape unit number (NT - Card III) must be specified for each of the cases of a particular run. Logical units 5 and 6 are used for the card reader and printer. Three scratch tapes are used by the code.

Utilizing FORTRAN IV language, it has been noted that double-precision arithmetic is necessary for operation on an IBM 360/65 computer. This additional word length is not believed necessary when the program is employed on machines with a longer built-in word length (such as a CDC 6600 machine).

The following limitation is placed upon the values of the Poisson's ratios of the materials:

$$\nu_{s\theta}\nu_{\theta s} < 1$$

If this criterion is violated, the stiffness matrix is no longer positive definite.

The slope, ϕ , at a node is considered positive if the tangent drawn in the direction of increasing node numbers is rotated counterclockwise from the positive z-axis. A negative slope exists if the tangent is

rotated clockwise from the positive z-axis. The magnitude of ϕ is thus limited to the range $-180^\circ \leq \phi \leq 180^\circ$. Considering Fig. 2 it should be noted that at nodes a and b the value of ϕ is positive while nodes c and d give rise to negative values of ϕ . Reversing the direction of node numbering therefore changes both the sign and the magnitude of the slope at a node.

Restrictions or limitations placed upon input geometry are:

1. The magnitude of ϕ must be in the range $-180^\circ \leq \phi \leq 180^\circ$.
2. A generating segment cannot have a change in slope greater than 180° (this restriction is easily circumvented by using a greater number of segments to generate the element data).
3. The generating segments must always be placed in the r - z coordinate system so that r and z are always positive (first quadrant).
4. The circular and parabolic segments cannot be degenerated to flat elements (linear segments).
5. A segment cannot contain, except at an end point, the "discontinuity" in slope at $\pm 180^\circ$. To circumvent this restriction use one segment whose slope is -180° at its final node and a second segment whose initial slope is $+180^\circ$.
6. Only elements which are consecutively numbered may intersect. This restriction is imposed to maintain the banding which is present in the stiffness and mass matrices.

In addition to the major limitations presented here, the section describing the input to the code enumerates a number of minor requirements which are necessary for valid input data. Other limitations and guidelines are also presented in the user's manuals for the solution programs: SNASOR II, DYNASOR II, and FAMSOR.

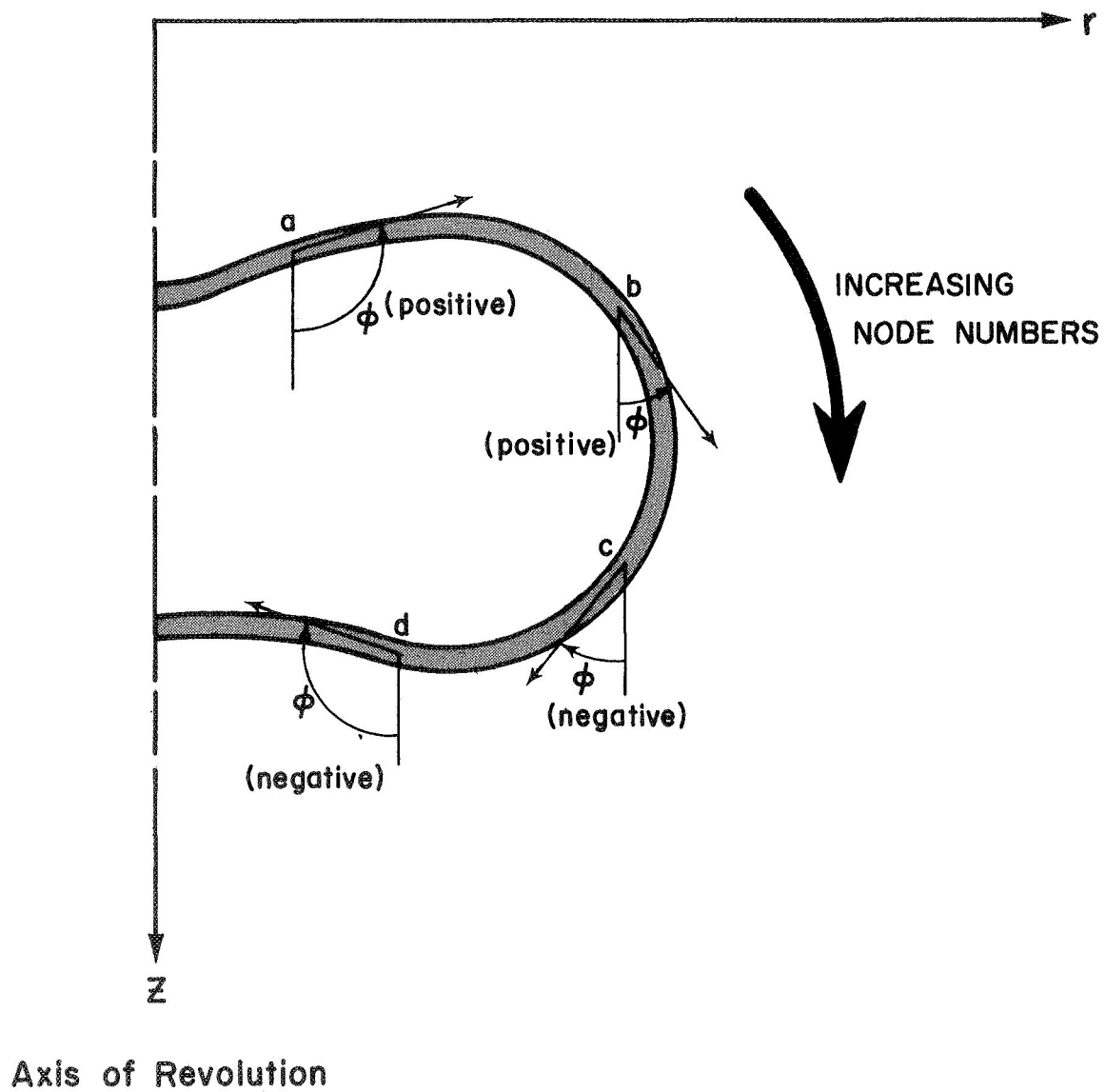


FIG.2 SIGN CONVENTION FOR SLOPE SPECIFICATION

SECTION III

Program Input

The SAMMSOR code has been written so that the program can readily be utilized by investigators who are not familiar with the inner workings of the program. Using the hints and adhering to the limitations presented in the previous section, it is believed that a user with only a superficial knowledge of the finite element method can effectively employ the SAMMSOR II code.

The double-precision version of the code requires a storage space of about 190K bytes on the IBM 360/65. The single-precision code requires approximately 110K bytes. The code is written in the FORTRAN IV language.

The input data for the program consists of one card I (card types will be explained on the following pages) followed by a complete data set for each of the cases to be run. A final card VIII is added at the end of the last data set. A case consists of cards of type II through VII and is all the input data required to generate the desired stiffness and mass matrices for a particular shell. The set of cards II - VII for a case is also referred to as a data set. There is no limit on the number of cases which can be included per run, but the logical unit number of the output tape (NT from Card II) must be different for each case of a run.

I. RUN CONTROL CARD

This card is used to identify the number of cases to be run and the logical unit numbers of the scratch tapes used in the run. (ONLY ONE CARD I IS USED PER RUN.)

Card Type I Format (4I5)		
Columns	Variable	Description
1-5	NCASES	The number of different data sets utilized for this run.
6-10	ND	Logical unit number for scratch tape. All the input data for the run is read onto this unit at the start of the run and is stored here until needed in the program.
11-15	NS	Logical unit number for scratch tape used by the program.
16-20	NS2	Logical unit number of a second scratch tape used by the code.

II. CASE IDENTIFICATION CARDS

These cards allow the user to print out comments which identify the problem being run.

A. Control Card (ONE CARD II-A PER DATA SET)

Card Type II-A Format (2I5)		
Columns	Variable	Description
1-5	NCARDS	Number of comment cards (Type II-B) which follow.
6-10	NT	Unit number of the tape on which stiffness and mass will be stored. The value of NT must be different for each case of a run.

B. Identification Cards

The information printed on these cards is printed as output for the SOR programs and should identify the particular shell being analyzed.
(IF NCARDS=0, OMIT CARDS II-B, OTHERWISE INCLUDE NCARDS OF TYPE II-B.)

Card Type II-B Format (20A4)		
Columns	Variable	Description
1-80	COMENT	Any desired alphanumeric information may be printed on these cards.

III. CASE CONTROL CARD

Identifies the number of elements and harmonics used and gives the unit number for the output tape. (ONE CARD III PER DATA SET)

Card Type III Format (3I5)		
Columns	Variable	Description
1-5	NELEMS	Total number of elements used to idealize the shell ($NELEMS \leq 50$).
6-10	IA	Total number of harmonics for which mass and stiffness matrices are to be calculated ($1 \leq IA \leq 20$).
11-15	NPRNMS	If the mass and stiffness matrices are to be printed, set NPRNMS = 1. If not, set NPRNMS = 0.

IV. MATERIAL PROPERTIES

The elastic and mass properties in the meridional and circumferential directions are constant over an element, but the properties can vary from element to element. (THE NUMBER OF CARDS OF TYPE IV $\leq$ NELEMS (CARD III) PER DATA SET.)

Card Type IV	Format (2I5, 5F10.0)	
Columns	Variable	Description
1-5	IELM1	Number of the first element to which the properties on this card apply.
6-10	IELM2	Number of the last element to which the properties on this card apply.
11-20	EE1	Young's Modulus of elasticity in the meridional direction (psi).
21-30	EE2	Young's Modulus of elasticity in the circumferential direction (psi).
31-40	GG	Shear modulus (psi). For an isotropic material, $GG=E/2(1+\nu)$
41-50	FFNU1	Poisson's ratio. The contraction in the meridional direction due to stress in the circumferential direction.
51-60	RHO	Density of the material (lb./in.^3).

NOTE: The elastic and mass properties can be read in for each individual element using NELEMS cards of type IV. If the properties of a consecutively numbered group of elements are the same, the input data can be simplified considerably by using only one card to describe the properties of this group of elements. In particular, if the elastic and mass properties of all the elements are the same, only one card of type IV is necessary per data set.

EXAMPLE: Consider a shell that is idealized using 20 elements. Assuming that the elastic properties of elements 1 through 10 are the same, different properties exist for the eleventh element and elements 12 through 20 have the same properties as the first group. Three cards should be

used to input the properties of the shell. The following values for IELM1 and IELM2 should be used.

Card 1:	IELM1 = 1	IELM2 = 10
Card 2:	IELM1 = 11	IELM2 = 11
Card 3:	IELM1 = 12	IELM2 = 20

A less sophisticated procedure would be to use one card for each element. Note that the properties for the elements 12-20 must be read in on a different card from those of elements 1-10.

V. THICKNESS

It is assumed that the thickness is a constant over the element in the meridional and the circumferential directions. Essentially the same procedure is used here as is employed with cards IV for the material properties. (THE NUMBER OF CARDS OF TYPE V MUST BE $\leq$ NELEMS FOR ANY GIVEN DATA SET.)

Card Type V Format (2I5, F10.0)		
Columns	Variable	Description
1-5	IELM1	Number of the first element to which thickness properties on this card apply.
6-10	IELM2	Number of the last element to which thickness properties on this card apply.
11-20	TT	Thickness of the shell element or group of elements (in).

VI. SHELL GEOMETRY

A most important part of any shell of revolution analysis is the generation of the geometry and the element idealization of the shell. For this reason, an extended discussion follows which describes in detail the procedure used to generate the geometry of the idealized shell. Slope discontinuities between elements are not allowed.

The shell nodal coordinates and slopes can be generated using either of two options available (or a combination of the two):

OPTION 1: The user may specify the coordinates and slopes of the nodes by inputting the values of these parameters at each of the nodes.

OPTION 2: For shells of relatively simple geometry (shallow caps, hemispheres, cylinders, etc.) the program has the capability of generating internally the desired nodal point data.

The shell is considered as being composed of one or more segments, with the segments being divided, in turn, into one or more sub-segments. The sub-segments are then divided into the desired number of finite elements. A segment may contain only one element or the entire shell may in some cases be considered as a single segment. The choice of the segment breakdown depends upon the option selected for inputting the shell geometry as well as the geometry of the shell itself.

If the geometry for a particular segment is to be generated internally, the profile of the segment must be linear, circular or parabolic. A segment is divided into a maximum of five sub-segments with each sub-segment being divided, in turn, into a specified number of equally spaced elements. Obviously, if the segment consists of only one element, no division into sub-segments is necessary. A set of cards (type VI) is required for each segment, with a set consisting of one card of types VI-A, VI-B, and VI-C -- in that order.

In order to generate the required nodal point data for a given shell, the following procedure must be used:

1. Select a breakdown of the shell into segments.
2. Establish the desired breakdown of the segments into sub-segments and determine the number of elements to be used for each sub-segment.
3. Input the required three data cards for each segment until the data for all the segments has been input.

The total number of data cards necessary for establishing the geometry of the entire shell is equal to three times the number of segments. If the first option is used for generating the geometry, the user will find that the total number of cards required will equal $(3 * NELEMS)$ where NELEMS was defined on card III. However, only $(NELEMS + 1)$ of these cards will be different since duplicate cards and blank cards are used. The procedure may therefore seem somewhat cumbersome when the user desires to specify the slopes and coordinates of each node. The procedure is, however, extremely simple when the nodal point data is to be internally generated. In addition, it has been found that fewer input errors arise when using this simple but repetitious procedure. It is for these reasons that the procedure is used in the code. An example of the effective use of this scheme can be seen by considering the first example problem presented where only three cards are required to generate the desired nodal data.

A. Segment initial node data

The data furnished by this card provides the node number, slope and coordinates of the initial node of a given segment.

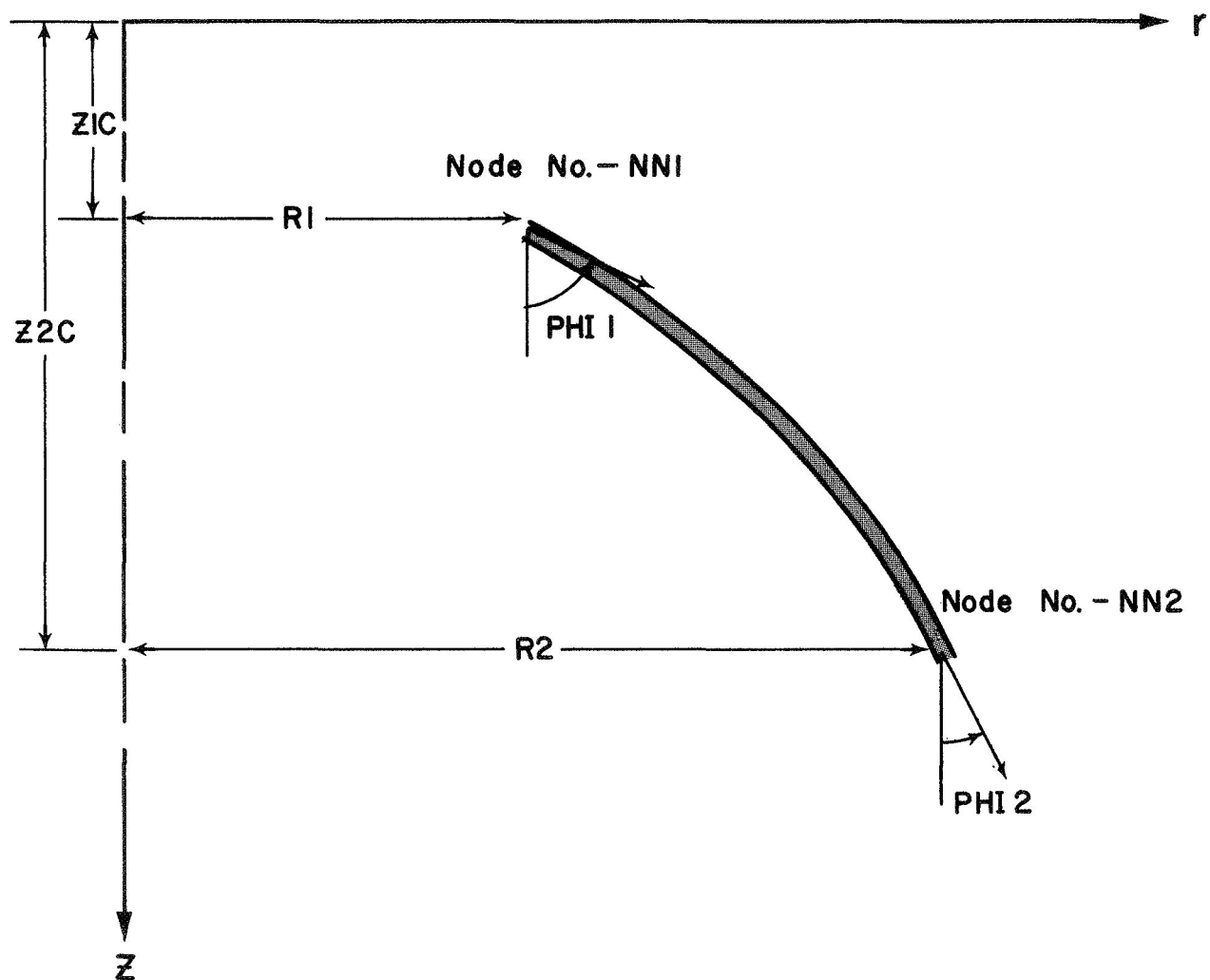
Card Type VI-A Format (I5, 3F10.0)		
Columns	Variable	Description
1-5	NN1	Number of the node at the beginning of this shell segment.
6-15	Z1C	Axial coordinate of node NN1.
16-25	R1	Radial coordinate of node NN1.
26-35	PHI1	Slope of the shell in the meridional direction at node NN1 (degrees).

***The slopes PHI1 and PHI2 are measured from the positive axis of revolution to the tangent to the meridian (see Fig. 2). Refer to Fig. 3 for an explanation of the input parameters.

B. Segment terminal node data

The data furnished by this card provides the node number, slope, and coordinates of the final node of a given segment.

Card Type VI-B Format (I5, 3F10.0)		
Columns	Variable	Description
1-5	NN2	Number of the node at the end of the segment (<u>NN2 must be > NN1</u>).
6-15	Z2C	Axial coordinate of node NN2.
16-25	R2	Radial coordinate of node NN2.
26-35	PHI2	Slope of the shell in the meridional direction at node NN2 (degrees).



Axis of Revolution

FIG. 3 INPUT VARIABLES SPECIFYING THE GEOMETRY OF A SHELL SEGMENT

C. Coordinate generation control card

This card controls the internal generation of data for the shell segment whose end coordinates are described on cards III-A and III-B of this set by describing the type of shell profile and the element breakdown to be used in the idealization of the segment. Each segment can be divided into a maximum of five sub-segments.

If ($NN2 = NN1 + 1$), obviously, the segment under consideration consists of only one element and the coordinates and slopes at the two adjacent nodes completely define the geometry of the shell segment. If this is the case, the user must include a card III-C in the set; but it must be blank.

Card Type VI-C Format (I5, 5(I5, F10.0))		
Columns	Variable	Description
1-5	ICLASS	Control parameter which defines the shell profile ICLASS = 1 - linear profile ICLASS = 2 - circular profile ICLASS = 3 - parabolic profile
6-10	NERAT (1)	This data specifies the number of elements and the relative lengths (decimal fractions) into which the sub-segments of the shell segment are divided. <u>The sum of the RA (I)'s must equal 1.0.</u>
11-20	RA (1)	
21-25	NERAT (2)	
26-35	RA (2)	
36-40	NERAT (3)	
41-50	RA (3)	
51-55	NERAT (4)	
56-65	RA (4)	
66-70	NERAT (5)	
71-80	RA (5)	

An understanding of the use of this card can be obtained by studying the input data for the example problems and by considering the following example. Consider a circular segment to be divided into four sub-segments which are each divided into five elements. Sub-segment lengths are

given as 1/10, 2/10, 1/2, and 2/10 respectively of the length of the segment. The following values would be used on card VI-C for this segment:

ICLASS = 2	(circular profile)
NERAT(1) = 5	RA(1) = 0.10
NERAT(2) = 5	RA(2) = 0.20
NERAT(3) = 5	RA(3) = 0.50
NERAT(4) = 5	RA(4) = 0.20
NERAT(5) = 0	RA(5) = <u>0.00</u>
	$\Sigma = 1.0$

If the segment was divided instead into 20 equally spaced elements, then the data would be:

ICLASS = 2	
NERAT(1) = 20	RA(1) = 1.0
NERAT(2) = 0	RA(2) = 0.0
NERAT(3) = 0	RA(3) = 0.0
NERAT(4) = 0	RA(4) = 0.0
NERAT(5) = 0	RA(5) = <u>0.0</u>
	$\Sigma = 1.0$

VII. FINAL DATA CARD FOR A CASE

Place this card after the last card VI-C of each data set. This signifies the end of the input data for a case. (USE ONE CARD VII PER DATA SET.)

Card Type VII	
Columns	Punch
1-11	END OF CASE

VIII. FINAL DATA CARD FOR A RUN

This card must be placed after the card VII of the last case to be run. This card denotes the end of the input data for a run. (ONLY ONE CARD VIII IS USED PER RUN.)

Card Type VIII	
Columns	Punch
1-10	END OF RUN

SECTION IV

EXAMPLE PROBLEMS

The example problems which follow were chosen to demonstrate the various facets of the input data required to run a given case and to acquaint the user with typical problems which have been analyzed using the SOR programs. As stated in a prior section, the choice of the element idealization must be judiciously selected. The example problems should provide the user with a "feel" for the selection of the element idealization for an analysis. The input data for the various shell geometries should answer any questions the user might have concerning the input procedure for cards VI of the input section.

Stiffness and mass matrices for selected harmonics are presented as output for the example problems. The output was generated using double-precision arithmetic on an IBM 360/65 system.

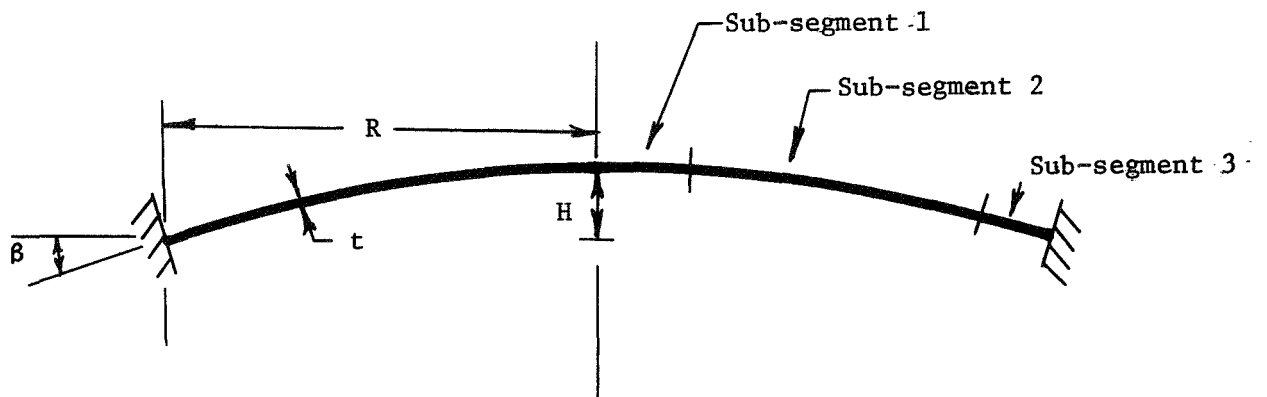
Example Problem 1

The first example problem is a shallow spherical cap ($\lambda = 6$) that has been analyzed extensively using the SNASOR II, DYNASOR II and the FAMSOR codes. The geometric and material properties for the shell are presented in Fig. 4.

The selection of the element breakdown and the number of harmonics to be generated depends not only upon the geometry of the shell, but also upon the loading to be applied to the shell. In order to be certain that the data for a sufficient number of harmonics will be generated, six (6) cosine harmonics will be used. The static and dynamic analyses previously conducted on this shell have shown that thirty (30) elements provide a good idealization for the loadings used providing these elements are concentrated near the apex of the shell, where a singularity obviously exists, and near the clamped edge.

A set of input data for this case is presented in Fig. 5. The geometry is input using the second option for inputting the data and the shell is considered as a single segment with a circular profile. Obviously, the geometry could have been input for each node, but the process would have been tedious.

A selected portion of the output generated for this case is presented in Figs. 6-10.



SHELL DESCRIPTION

$R = 0.9$ in.

$H = 0.0859$ in.

$t = 0.01576$ in.

$\lambda = 6.0$

$\beta = 10.9^\circ$

MATERIAL PROPERTIES
ALUMINUM

$E = 10.0 \times 10^6$ psi

$G = 3.85 \times 10^6$ psi

$\nu = 0.30$

$\rho = 0.0942$ lb./in.³

ELEMENT BREAKDOWN

Sub-segment 1 - 6 elements

Sub-segment 2 - 12 elements

Sub-segment 3 - 12 elements

Fig. 4 SHALLOW SPHERICAL CAP

GEOMETRIC AND ELASTIC PROPERTIES OF STRUCTURE												
ELEM. NO.	NODE NOS.	COORDINATES Z	R	SLOPE	S	THICKNESS	E1	ELASTIC CONSTANTS E2	G	NU1	RHO	
1	1	0.0	0.100000-05	0.900000 02	0.15090-01	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	2	0.956970-04	0.301820-01	0.896370 02								
2	2	0.956970-04	0.301820-01	0.896370 02	0.45270-01	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	3	0.382780-03	0.603620-01	0.892730 02								
3	3	0.382780-03	0.603620-01	0.892730 02	0.75450-01	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	4	0.861250-03	0.905400-01	0.889100 02								
4	4	0.861250-03	0.905400-01	0.889100 02	0.10560 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	5	0.153110-02	0.120710 00	0.885470 02								
5	5	0.153110-02	0.120710 00	0.885470 02	0.13580 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	6	0.239220-02	0.150880 00	0.881830 02								
6	6	0.239220-02	0.150880 00	0.881830 02	0.16600 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	7	0.344470-02	0.181050 00	0.878200 02								
7	7	0.344470-02	0.181050 00	0.878200 02	0.20560 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	8	0.556280-02	0.230050 00	0.872300 02								
8	8	0.556280-02	0.230050 00	0.872300 02	0.25470 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	9	0.818570-02	0.279020 00	0.866390 02								
9	9	0.818570-02	0.279020 00	0.866390 02	0.30370 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	10	0.113130-01	0.327970 00	0.860490 02								
10	10	0.113130-01	0.327970 00	0.860490 02	0.35270 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	11	0.149450-01	0.376880 00	0.854580 02								
11	11	0.149450-01	0.376880 00	0.854580 02	0.40180 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	12	0.190800-01	0.425750 00	0.848680 02								
12	12	0.190800-01	0.425750 00	0.848680 02	0.45080 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	13	0.237190-01	0.474570 00	0.842770 02								
13	13	0.237190-01	0.474570 00	0.842770 02	0.49990 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	14	0.288610-01	0.523350 00	0.836870 02								
14	14	0.288610-01	0.523350 00	0.836870 02	0.54890 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	15	0.345050-01	0.572070 00	0.830970 02								
15	15	0.345050-01	0.572070 00	0.830970 02	0.59800 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	16	0.406510-01	0.620730 00	0.825060 02								
16	16	0.406510-01	0.620730 00	0.825060 02	0.64700 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	17	0.472970-01	0.665320 00	0.819160 02								
17	17	0.472970-01	0.665320 00	0.819160 02	0.69610 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	18	0.544450-01	0.717840 00	0.813250 02								
18	18	0.544450-01	0.717840 00	0.813250 02	0.74510 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	19	0.620910-01	0.766290 00	0.807350 02								
19	19	0.620910-01	0.766290 00	0.807350 02	0.77530 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	20	0.639270-01	0.777460 00	0.805990 02								
20	20	0.639270-01	0.777460 00	0.805990 02	0.78660 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	21	0.657890-01	0.788620 00	0.804620 02								
21	21	0.657890-01	0.788620 00	0.804620 02	0.79790 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	22	0.676780-01	0.795780 00	0.803260 02								
22	22	0.676780-01	0.795780 00	0.803260 02	0.80920 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	23	0.695930-01	0.810930 00	0.801900 02								
23	23	0.695930-01	0.810930 00	0.801900 02	0.82060 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	24	0.715340-01	0.822080 00	0.800540 02								
24	24	0.715340-01	0.822080 00	0.800540 02	0.83190 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	25	0.735030-01	0.833230 00	0.799170 02								
25	25	0.735030-01	0.833230 00	0.799170 02	0.84320 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	26	0.754970-01	0.844370 00	0.797810 02								
26	26	0.754970-01	0.844370 00	0.797810 02	0.85450 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	27	0.775190-01	0.855510 00	0.796450 02								
27	27	0.775190-01	0.855510 00	0.796450 02	0.86580 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	28	0.795660-01	0.866640 00	0.795090 02								
28	28	0.795660-01	0.866640 00	0.795090 02	0.87720 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	29	0.816400-01	0.877760 00	0.793720 02								
29	29	0.816400-01	0.877760 00	0.793720 02	0.88850 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	30	0.837410-01	0.888890 00	0.792360 02								
30	30	0.837410-01	0.888890 00	0.792360 02	0.89980 00	0.15760-01	0.1000 08	0.1000 08	0.3850 07	0.300	0.09420	
	31	0.858680-01	0.900000 00	0.791000 02								

Fig. 6 ELEMENT PROPERTIES (OUTPUT) EXAMPLE PROBLEM 1

HARMONIC NUMBER		0 HAS THE FOLLOWING STIFFNESS MATRIX	
0.222543330	06	0.0	0.0
-0.854592700	02	0.0	0.0
-0.117313360	04	0.0	0.0
-0.222543330	06	0.0	0.0
-0.138475270	04	0.0	0.0
-0.335820300	04	0.0	0.0
-0.465254360	06	0.0	0.0
-0.126611500	05	0.0	0.0
-0.752790220	04	0.0	0.0
-0.754148090	06	0.0	0.0
-0.336748290	05	0.0	0.0
-0.119726450	05	0.0	0.0
-0.104822410	07	0.0	0.0
-0.647659730	05	0.0	0.0
-0.164343690	05	0.0	0.0
-0.134446390	07	0.0	0.0
-0.105887390	06	0.0	0.0
-0.208987180	05	0.0	0.0
-0.164220110	07	0.0	0.0
-0.157013360	06	0.0	0.0
-0.253617910	05	0.0	0.0
-0.481209690	06	0.0	0.0
-0.183019040	06	0.0	0.0
-0.119706840	05	0.0	0.0
-0.599774420	06	0.0	0.0
-0.278665650	06	0.0	0.0
-0.147303410	05	0.0	0.0
-0.721657860	06	0.0	0.0
-0.394143230	06	0.0	0.0
-0.174574960	05	0.0	0.0
-0.847388660	06	0.0	0.0
-0.529326140	06	0.0	0.0
-0.201768020	05	0.0	0.0
-0.977525930	06	0.0	0.0
-0.684070510	06	0.0	0.0
-0.228063160	05	0.0	0.0
-0.111263970	07	0.0	0.0
-0.598212360	06	0.0	0.0
-0.259553810	05	0.0	0.0
-0.125330280	07	0.0	0.0
-0.105156740	07	0.0	0.0
-0.282725910	05	0.0	0.0
-0.140008610	07	0.0	0.0
-0.126393130	07	0.0	0.0
-0.309467640	05	0.0	0.0
-0.155356900	07	0.0	0.0
-0.149507940	07	0.0	0.0
-0.336057400	05	0.0	0.0
-0.171427660	07	0.0	0.0
-0.174676700	07	0.0	0.0
-0.362513700	05	0.0	0.0
-0.188279990	07	0.0	0.0
-0.201272990	07	0.0	0.0
-0.388795190	05	0.0	0.0
-0.205967370	07	0.0	0.0
-0.229868410	07	0.0	0.0
-0.414900610	05	0.0	0.0
-0.142057420	09	0.0	0.0
-0.111239590	08	0.0	0.0
-0.805395610	06	0.0	0.0
-0.144058210	09	0.0	0.0
-0.114439980	08	0.0	0.0
-0.816695710	06	0.0	0.0
-0.144058700	09	0.0	0.0
-0.117682460	08	0.0	0.0
-0.827977360	06	0.0	0.0
-0.148050360	09	0.0	0.0
-0.120946430	08	0.0	0.0
-0.839240800	06	0.0	0.0
-0.150041650	09	0.0	0.0
-0.124292960	08	0.0	0.0
-0.850484220	06	0.0	0.0
-0.152029690	09	0.0	0.0
-0.127660610	08	0.0	0.0
-0.861708940	06	0.0	0.0
-0.154014430	09	0.0	0.0
-0.131069590	08	0.0	0.0
-0.872914180	06	0.0	0.0
-0.155995840	09	0.0	0.0
-0.134514720	08	0.0	0.0
-0.884099670	06	0.0	0.0
-0.157973860	09	0.0	0.0
-0.138010810	08	0.0	0.0
-0.895265180	06	0.0	0.0
-0.159948510	09	0.0	0.0
-0.141542650	08	0.0	0.0
-0.906410450	06	0.0	0.0
-0.161919680	09	0.0	0.0
-0.145115040	08	0.0	0.0
-0.917535220	06	0.0	0.0
-0.163887350	09	0.0	0.0
-0.148727770	08	0.0	0.0
-0.928639250	06	0.0	0.0
0.392402210	09	0.0	0.0
0.228535660	01	0.0	0.0
0.854592700	02	0.0	0.0
0.117313360	04	0.0	0.0
0.138475270	04	0.0	0.0
0.213225180	01	0.0	0.0
0.911536230	04	0.0	0.0
0.147056490	07	0.0	0.0
0.920976300	02	0.0	0.0
0.282325730	05	0.0	0.0
0.261110150	07	0.0	0.0
0.233709530	03	0.0	0.0
0.573257850	05	0.0	0.0
0.372919460	07	0.0	0.0
0.431607270	03	0.0	0.0
0.964240890	05	0.0	0.0
0.1058275310	07	0.0	0.0
0.686080770	03	0.0	0.0
0.145550760	06	0.0	0.0
0.592897350	07	0.0	0.0
0.977164930	03	0.0	0.0
0.168623020	06	0.0	0.0
0.448805950	07	0.0	0.0
0.741194680	03	0.0	0.0
0.260977420	06	0.0	0.0
0.278665550	06	0.0	0.0
0.107145470	04	0.0	0.0
0.373125006	06	0.0	0.0
0.145796250	04	0.0	0.0
0.666530340	07	0.0	0.0
0.504979550	06	0.0	0.0
0.774311010	07	0.0	0.0
0.190058360	04	0.0	0.0
0.656352110	06	0.0	0.0
0.881416510	07	0.0	0.0
0.339914510	04	0.0	0.0
0.827200060	06	0.0	0.0
0.987823750	07	0.0	0.0
0.295346360	04	0.0	0.0
0.101722130	07	0.0	0.0
0.109349110	08	0.0	0.0
0.356324920	04	0.0	0.0
0.122625250	07	0.0	0.0
0.119834700	08	0.0	0.0
0.422830560	04	0.0	0.0
0.145407020	07	0.0	0.0
0.328840530	05	0.0	0.0
0.130239430	08	0.0	0.0
0.494033270	04	0.0	0.0
0.170043040	07	0.0	0.0
0.140551270	08	0.0	0.0
0.572302620	04	0.0	0.0
0.196506940	07	0.0	0.0
0.150764020	08	0.0	0.0
0.655205800	04	0.0	0.0
0.224770390	07	0.0	0.0
0.160877410	08	0.0	0.0
0.743507700	04	0.0	0.0
0.141205740	09	0.0	0.0
0.111239590	08	0.0	0.0
0.805395610	06	0.0	0.0
0.144058210	09	0.0	0.0
0.114439980	08	0.0	0.0
0.816695710	06	0.0	0.0
0.144058700	09	0.0	0.0
0.117682460	08	0.0	0.0
0.827977360	06	0.0	0.0
0.148050360	09	0.0	0.0
0.120946430	08	0.0	0.0
0.839240800	06	0.0	0.0
0.150041650	09	0.0	0.0
0.124292960	08	0.0	0.0
0.850484220	06	0.0	0.0
0.152029690	09	0.0	0.0
0.127660610	08	0.0	0.0
0.861708940	06	0.0	0.0
0.154014430	09	0.0	0.0
0.131069590	08	0.0	0.0
0.872914180	06	0.0	0.0
0.155995840	09	0.0	0.0
0.134514720	08	0.0	0.0
0.884099670	06	0.0	0.0
0.157973860	09	0.0	0.0
0.138010810	08	0.0	0.0
0.895265180	06	0.0	0.0
0.159948510	09	0.0	0.0
0.141542650	08	0.0	0.0
0.906410450	06	0.0	0.0
0.161919680	09	0.0	0.0
0.145115040	08	0.0	0.0
0.917535220	06	0.0	0.0
0.163887350	09	0.0	0.0
0.148727770	08	0.0	0.0
0.928639250	06	0.0	0.0
0.142140870	08	0.0	0.0
0.902653720	06	0.0	0.0
0.1680497430	08	0.0	0.0
0.169210060	06	0.0	0.0
0.145720690	09	0.0	0.0
0.913784230	06	0.0	0.0
0.1477667490	08	0.0	0.0
0.173558860	06	0.0	0.0
0.149341250	08	0.0	0.0
0.924894060	06	0.0	0.0
0.163744300	08	0.0	0.0
0.342702190	06	0.0	0.0
0.131644900	08	0.0	0.0
0.869140680	06	0.0	0.0
0.155367490	08	0.0	0.0
0.333437920	06	0.0	0.0
0.880331700	06	0.0	0.0
0.159529300	08	0.0	0.0
0.338270960	06	0.0	0.0
0.891502800	06	0.0	0.0
0.163744300	08	0.0	0.0
0.342702190	06	0.0	0.0
0.142140870	08	0.0	0.0
0.902653720	06	0.0	0.0
0.1680497430	08	0.0	0.0
0.169210060	06	0.0	0.0
0.145720690	09	0.0	0.0
0.913784230	06	0.0	0.0
0.1477667490	08	0.0	0.0
0.173558860	06	0.0	0.0
0.149341250	08	0.0	0.0
0.924894060	06	0.0	0.0
0.163744300	08	0.0	0.0
0.342702190	06	0.0	0.0
0.131644900	08	0.0	0.0
0.869140680	06	0.0	0.0
0.155367490	08	0.0	0.0
0.333437920	06	0.0	0.0
0.880331700	06	0.0	0.0
0.159529300	08	0.0	0.0
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0.142140870	08	0.0	0.0
0.902653720	06	0.0	0.0
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0.169210060	06	0.0	0.0
0.145720690	09	0.0	0.0
0.913784230	06	0.0	0.0
0.1477667490	08	0.0	0.0
0.173558860	06	0.0	0.0
0.149341250	08	0.0	0.0
0.924894060	06	0.0	0.0
0.163744300	08	0.0	0.0
0.342702190	06	0.0	0.0
0.131644900	08	0.0	0.0
0.869140680	06	0.0	0.0
0.155367490	08	0.0	0.0
0.333437920	06	0.0	0.0
0.880331700	06	0.0	0.0
0.159529300	08	0.0	0.0
0.338270960	06	0.0	0.0
0.891502800	06	0.0	0.0
0.163744300	08	0.0	0.0
0.342702190	06	0.0	0.0
0.142140870	08	0.0	0.0
0.902653720	06	0.0	0.0
0.1680497430	08	0.0	0.0
0.169210060	06	0.0	0.0
0.145720690	09	0.0	0.0
0.913784230	06	0.0	0.0
0.1477667490	08	0.0	0.0
0.173558860	06	0.0	0.0
0.149341250	08	0.0	0.0
0.924894060	06	0.0	0.0
0.163744300	08	0.0	0.0
0.342702190	06	0.0	0.0
0.131644900	08	0.0	0.0
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0.155367490	08	0.0	0.0
0.333437920	06	0.0	0.0
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0.159529300	08	0.0	0.0
0.338270960	06	0.0	0.0
0.8			

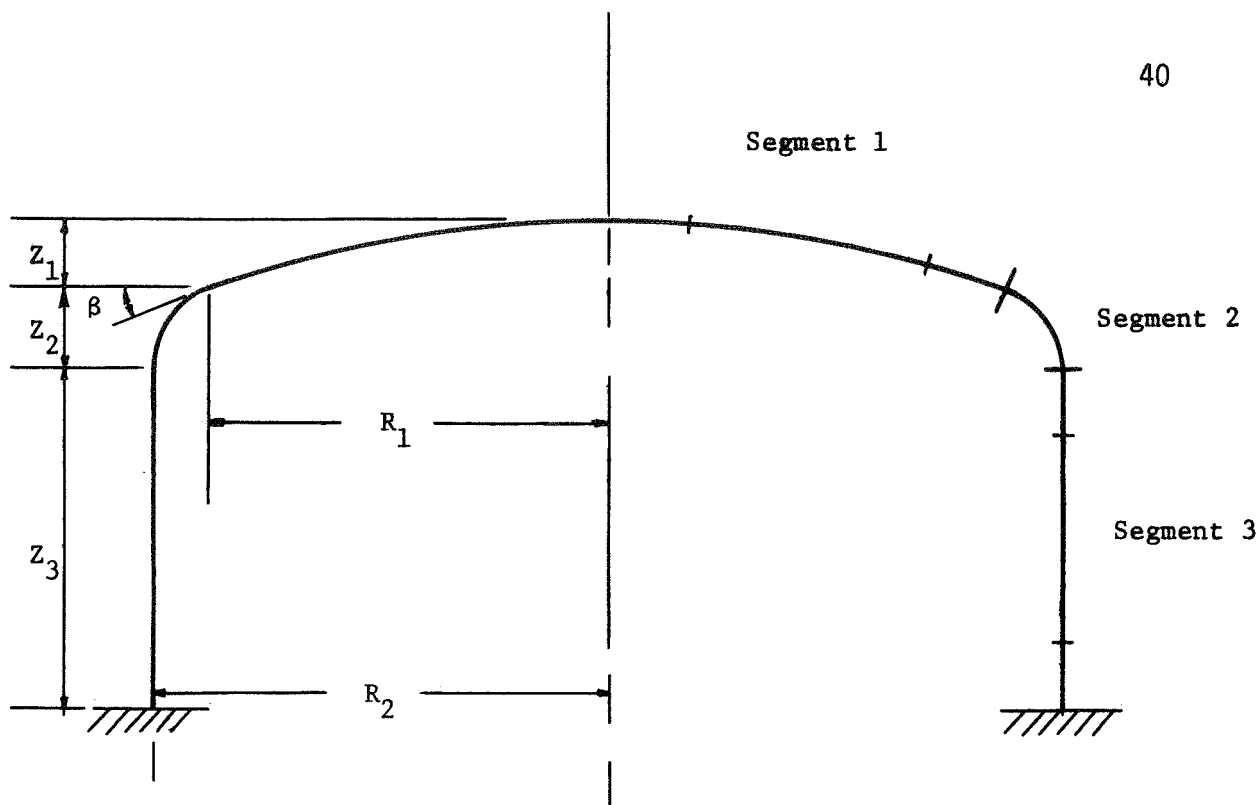
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-0.327777330-11	0.0	0.102550870-10	-0.956086160-11	0.175463560-07	0.0
-0.125800010-10	0.0	0.0	0.0	0.0	0.0
0.111472040-08	0.0	0.183355700-08	0.503735130-13	-0.247772890-10	0.0
0.0	0.0	0.150438750-13	0.751852280-13	-0.251594710-10	0.0
-0.602677440-11	0.0	0.447788520-10	-0.26604760-10	0.35055640-07	0.0
0.948858220-11	0.0	0.0	0.0	0.0	0.0
0.334434960-08	0.0	0.50161970-08	0.327304450-12	-0.795157840-10	0.0
0.0	0.0	-0.221316450-12	-0.237566730-12	-0.251553720-10	0.0
-0.406617450-11	0.0	0.975362450-10	-0.476526260-10	0.526398440-07	0.0
0.289379850-10	0.0	0.0	0.0	0.0	0.0
0.557434440-08	0.0	0.16854260-08	0.645788860-12	-0.170689700-09	0.0
0.0	0.0	-0.669160080-12	-0.395933940-12	-0.251456670-10	0.0
0.161300440-10	0.0	0.128342280-07	0.160574420-11	-0.298281850-09	0.0
0.475829280-10	0.0	-0.135850460-11	-0.354285220-12	-0.251346020-10	0.0
0.780498330-08	0.0	0.257683670-09	-0.457115450-10	0.877144200-07	0.0
0.545476700-10	0.0	0.164981850-07	0.246704810-11	-0.446240920-09	0.0
0.666206000-10	0.0	-0.228923690-11	-0.712614220-12	-0.251203790-10	0.0
0.100365220-07	0.0	0.365021090-09	-0.164721190-09	0.137587450-06	0.0
0.111166240-09	0.0	0.0	0.0	0.0	0.0
0.856481180-10	0.0	0.201599180-07	0.384953660-11	-0.812566470-09	0.0
0.122692160-07	0.0	-0.346120790-11	-0.870914960-12	-0.404555710-09	0.0
0.185956250-09	0.0	0.800926910-09	-0.361240750-09	0.208055960-06	0.0
0.104662240-09	0.0	0.05656960-07	0.146597450-10	-0.129729220-08	0.0
0.288204880-07	0.0	-0.144230480-10	-0.435273820-11	-0.995478650-10	0.0
0.215317800-09	0.0	0.114176450-08	-0.446581460-09	0.252314130-06	0.0
0.357002810-09	0.0	0.502239950-07	0.252068520-10	-0.189455990-08	0.0
0.357002810-09	0.0	-0.224389720-10	-0.539011060-11	-0.493754490-10	0.0
0.357002810-09	0.0	0.153972050-08	-0.531727900-09	0.296525210-06	0.0
0.357002810-09	0.0	0.598636930-07	0.355108420-10	-0.260979980-08	0.0
0.357002810-09	0.0	-0.322119560-10	-0.642691060-11	-0.991699930-10	0.0
0.357002810-09	0.0	0.159944190-08	-0.616643920-09	0.340680970-06	0.0
0.357002810-09	0.0	0.0	0.0	0.0	0.0
0.357002810-09	0.0	0.056480010-07	0.475673600-10	-0.143425420-08	0.0
0.357002810-09	0.0	-0.437393480-10	-0.746302810-11	-0.969315840-10	0.0
0.357002810-09	0.0	0.250551210-08	-0.701293450-09	0.384773250-06	0.0
0.357002810-09	0.0	0.790891430-07	0.613712250-10	-0.437504440-08	0.0
0.357002810-09	0.0	-0.570157520-10	-0.849835320-11	-0.986603230-10	0.0
0.357002810-09	0.0	0.307245460-08	-0.785640520-09	0.428793880-06	0.0
0.357002810-09	0.0	0.886673580-07	0.769166320-10	-0.542517340-08	0.0
0.357002810-09	0.0	-0.720355290-10	-0.953277590-11	-0.983563260-10	0.0
0.357002810-09	0.0	0.364973320-08	-0.869649330-09	0.472734750-06	0.0
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0.357002810-09	0.0	-0.887923010-10	-0.105661860-10	-0.980197210-10	0.0
0.357002810-09	0.0	0.437175290-08	-0.953284180-09	0.516577780-06	0.0
0.357002810-09	0.0	0.107454040-06	0.113204870-09	-0.785486170-08	0.0
0.357002810-09	0.0	-0.107278950-09	-0.115984750-10	-0.975050520-10	0.0
0.357002810-09	0.0	0.510285980-08	-0.103650960-08	0.560344950-06	0.0
0.357002810-09	0.0	0.117240080-06	0.133932290-09	-0.923183670-08	0.0
0.357002810-09	0.0	-0.127487620-09	-0.126295320-10	-0.972492760-10	0.0
0.357002810-09	0.0	0.588734250-08	-0.111929010-08	0.603998280-06	0.0
0.357002810-09	0.0	0.124703320-06	0.156370410-09	-0.107149830-07	0.0
0.357002810-09	0.0	-0.149405760-09	-0.136592470-10	-0.968157620-10	0.0
0.357002810-09	0.0	0.672443220-08	-0.120159070-08	0.647539840-06	0.0
0.357002810-09	0.0	0.136124110-06	0.180509710-09	-0.123027220-07	0.0
0.357002810-09	0.0	-0.17935980-09	-0.146875130-10	-0.963502950-10	0.0
0.357002810-09	0.0	0.761330370-08	-0.128337630-08	0.458171170-06	0.0
0.357002810-09	0.0	0.145508810-06	0.206339920-09	-0.142606870-07	0.0
0.357002810-09	0.0	-0.198356320-09	-0.157142190-10	0.213055310-08	0.0
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0.357002810-09	0.0	0.138940600-07	0.573945200-11	-0.158963530-07	0.0
0.357002810-09	0.0	-0.572324760-11	-0.338730620-12	-0.220530030-11	0.0
0.357002810-09	0.0	0.512010180-08	-0.351204420-10	0.241065380-06	0.0
0.357002810-09	0.0	0.343412400-07	0.590658740-11	-0.163496490-07	0.0
0.357002810-09	0.0	-0.589015500-11	-0.343631010-12	-0.220257370-11	0.0
0.357002810-09	0.0	0.343412400-07	0.590658740-11	-0.163496490-07	0.0
0.357002810-09	0.0	-0.589015500-11	-0.343631010-12	-0.220257370-11	0.0
0.357002810-09	0.0	0.343412400-07	0.590658740-11	-0.163496490-07	0.0
0.357002810-09	0.0	-0.589015500-11	-0.343631010-12	-0.220257370-11	0.0
0.357002810-09	0.0	0.343412400-07	0.590658740-11	-0.163496490-07	0.0
0.357002810-09	0.0	-0.589015500-11	-0.343631010-12	-0.220257370-11	0.0
0.357002810-09	0.0	0.343412400-07	0.590658740-11	-0.163496490-07	0.0
0.357002810-09	0.0	-0.589015500-11	-0.343631010-12	-0.220257370-11	0.0
0.357002810-09	0.0	0.343412400-07	0.590658740-11	-0.163496490-07	0.0
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0.357002810-09	0.0	-0.589015500-11	-0.343631010-12	-0.220257370-11	0.0
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0.357002810-09	0.0	-0.589015500-11	-0.343631010-12	-0.220257370-11	0.0
0.357002810-09	0.0	0.343412400-07	0.590658740-11	-0.163496490-07	0.0
0.357002810-09	0.0	-0.589015500-11	-0.343631010-12	-0.220257370-11	0.0
0.357002810-09	0.0	0.343412400-07	0.590658740-11	-0.163496490-07	0.0
0.357002810-09	0.0	-0.589015500-11	-0.343631010-12	-0.220257370-11	0.0
0.357002810-09	0.0	0.343412400-07	0.590658740-11	-0.163496490-07	0.0
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0.357002810-09	0.0	0.343412400-07	0.590658740-11	-0.163496490-07	0.0
0.357002810-09	0.0	-0.589015500-11	-0.343631010-12	-0.220257370-11	0.0
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0.357002810-09	0.0	0.343412400-07	0.590658740-11	-0.163496490-07	0.0
0.357002810-09	0.0	-0.589015500-11	-0.343631010-12	-0.220257370-11	0.0
0.357002810-09	0.0	0.343412400-07	0.590658740-11	-0.163496490-07	0.0
0.357002810-09	0.0	-0.589015500-11	-0.343631010-12	-0.220257370-11	0.0
0.357002810-09	0.0	0.343412400-07	0.590658740-11	-0.163496490-07	0.0
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0.357002810-09	0.0	0.343412400-07	0.590658740-11	-0.163496490-07	0.0
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0.357002810-09	0.0	0.343412400-07	0.590658740-11	-0.163496490-07	0.0
0.357002810-09	0.0	-0.589015500-11	-0.343631010-12	-0.220257370-11	0.0
0.357002810-09	0.0	0.343412400-07	0.590658740-11	-0.163496490-07	0.0
0.357002810-09	0.0	-0.589015500-11	-0.343631010-12	-0.220257370-11	0.0
0.357002810-09	0.0	0.343412400-07	0.590658740-11	-0.163496490-07	0.0
0.357002810-09	0.0	-0.589015500-11	-0.343631010-12	-0.220257370-11	0.0
0.357002810-09	0.0	0.343412400-07	0.590658740-11	-0.163496490-07	0.0
0.357002810-09	0.0	-0.589015500-11	-0.343631010-12	-0.220257370-11	0.0
0.357002810-09	0.0	0.343412400-07	0.590658740-11	-0.163496490-07	0.0
0.357002810-09	0.0	-0.589015500-11	-0.343631010-12	-0.220257370-11	0.0
0.357002810-09	0.0	0.343412400-07	0.590658740-11	-0.163496490-07	0.0
0.357002810-09	0.0	-0.589015500-11	-0.343631010-12	-0.220257370-11	0.0
0.357002810-09	0.0	0.343412400-07	0.590658740-11	-0.163496490-07	0.0
0.357002810-09	0.0	-0.589015500-11	-0.343631010-12	-0.220257370-11	0.0</

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0.59925617D 09	0.26475901D 09	0.26467129D 09	0.97387555D 04	0.37443739D 06	
0.97165701D 16	0.59944753D 03	0.60177931D 03	0.97387555D 04	0.11114365D 04	
-0.83915425D 07	0.23012549D 03	0.54948470D 02	0.11114365D 04	0.46656034D 02	
0.16072308D 03	-0.17055555D 04	0.13448059D 04	0.22877404D 01	0.76923879D 06	
0.10291256D 05	0.00335223D 05	0.70933264D 05	0.46605270D 00	0.46219272D 04	0.11225720D 07
-0.86430820D 06	0.11719820D 01	0.42178200D 01	0.11719820D 01	0.12124800D 04	-0.19311449D 01
-0.25429500D 06	0.18472539D 04	0.40578533D 04	0.36424570D 04	0.63781300D 06	0.99358215D 02
-0.12391453D 04	-0.15811530D 06	-0.80922811D 05	0.46370491D 01	0.37290540D 03	0.97587736D 06
-0.35250780D 07	0.21371205D 06	-0.64678532D 06	-0.36687420D 01	0.19974408D 06	0.22641940D 06
-0.38426790D 04	-0.53282476D 01	0.45716363D 02	0.33045563D 02	-0.14367097D 04	-0.21792721D 01
-0.38932380D 06	0.20786110D 04	-0.13494918D 05	0.52608702D 04	0.92055479D 06	0.12852009D 07
-0.23539416D 06	-0.21751612D 06	-0.11987229D 06	0.55252147D 01	0.44248772D 03	0.33503340D 07
-0.16050337D 05	0.21063131D 06	-0.12756670D 07	-0.51374590D 02	0.45143856D 05	0.15947236D 06
-0.60265034D 04	-0.58389890D 01	0.11600073D 03	0.53958866D 02	-0.14675604D 04	-0.40121952D 01
-0.53271762D 04	0.42427126D 04	-0.62006279D 05	0.74889670D 04	0.12110160D 07	0.16304833D 06
-0.34810034D 06	-0.42362279D 06	-0.13714703D 06	0.57249670D 01	0.46377359D 03	0.22064194D 06
-0.31605043D 05	0.20410410D 06	-0.18416870D 07	-0.12236753D 03	0.80277446D 05	0.12093017D 06
-0.82423932D 04	-0.59545941D 01	0.21568454D 03	0.78901231D 02	-0.14783844D 04	-0.12851870D 01
-0.67887441D 06	0.53658777D 04	-0.47532779D 06	0.97138860D 04	0.15051150D 07	0.44184900D 07
-0.48400731D 06	-0.62339131D 06	-0.14655180D 06	0.58172877D 01	0.47623791D 03	0.19902031D 07
-0.52172412D 05	0.19906811D 06	-0.23980228D 07	-0.22173700D 03	0.12533913D 06	0.97202460D 05
-0.10468184D 05	-0.59897020D 01	0.34302183D 03	0.10108742D 03	-0.14633590D 04	-0.10468043D 01
-0.82650960D 06	0.45201613D 05	-0.72626649D 05	0.11951161D 05	0.10696347D 07	0.19331142D 07
-0.57361867D 04	-0.16199542D 07	-0.15240440D 06	0.59683025D 01	0.52605916D 03	0.54917589D 07
-0.77740600D 05	0.19906811D 06	-0.23980228D 07	-0.22173700D 03	0.12533913D 06	0.97202460D 05
-0.12695210D 05	-0.61624050D 01	0.49586575D 03	0.12349394D 03	0.71488890D 06	-0.37794510D 01
-0.24327465D 06	-0.51571985D 04	-0.42393723D 03	0.55593153D 04	0.54524266D 06	0.18780460D 07
-0.69606735D 06	-0.76221050D 05	-0.14407352D 06	0.14273436D 02	0.82745872D 03	0.10340490D 06
-0.90655661D 03	0.20687950D 06	-0.22748211D 07	-0.10452704D 03	0.22187911D 06	0.51523488D 07
-0.60108422D 06	-0.13630874D 02	0.36385320D 03	0.96113904D 02	0.92041123D 03	-0.16699465D 03
-0.30201580D 06	0.10001348D 05	-0.12954569D 06	0.69316367D 04	0.66642710D 06	0.22427842D 07
-0.87858190D 04	-0.15543821D 06	-0.12071731D 06	0.14364795D 02	0.83077265D 03	0.18340897D 01
-0.13826610D 06	0.19705131D 05	-0.27742455D 07	-0.20508831D 03	0.32580197D 06	0.85537732D 05
-0.73570732D 04	-0.114472667D 02	0.33520272D 01	0.11665969D 03	-0.92625341D 03	-0.20295788D 03
-0.36259117D 06	0.11836111D 05	-0.18561233D 06	0.83011636D 04	0.78788617D 06	0.26120935D 07
-0.10612915D 06	-0.11544420D 07	-0.15508011D 06	0.14444864D 02	0.64235810D 03	0.72877003D 07
-0.19063070D 06	0.19906811D 06	-0.23980228D 07	-0.22173700D 03	0.12533913D 06	0.97202460D 05
-0.87367054D 04	-0.14523076D 02	0.37859157D 03	0.13917529D 03	-0.43200020D 03	-0.15724476D 01
-0.42519059D 06	0.13064594D 05	-0.25153165D 06	0.96688077D 04	0.91370040D 06	0.29840457D 07
-0.12437472D 05	-0.13680949D 07	-0.15812914D 06	0.14488895D 02	0.84540964D 03	0.83471247D 07
-0.28362810D 06	0.19180333D 06	-0.38598014D 07	-0.49145041D 03	0.59278534D 06	0.83471247D 07
-0.10095289D 05	-0.14573470D 02	0.95006211D 03	0.16168702D 03	0.93805323D 06	-0.13745747D 01
-0.49005627D 06	-0.54684190D 05	-0.32723210D 06	0.11033223D 05	0.10476620D 07	0.33575878D 07
-0.14256780D 05	-0.51506976D 07	-0.31603524D 06	0.14515929D 04	0.64715180D 03	0.50620517D 05
-0.34100279D 06	0.18996350D 06	-0.43967380D 07	-0.67703053D 03	0.75556688D 06	0.94107786D 07
-0.11445710D 05	-0.14572570D 02	0.11993653D 03	0.18413542D 03	-0.94459347D 03	-0.12200543D 01
-0.55749524D 06	0.17310722D 05	-0.41263212D 06	0.12393735D 05	0.11852852D 07	0.37320875D 07
-0.16079570D 05	-0.17320666D 07	-0.16202102D 06	0.14535045D 02	0.84806814D 03	0.50433837D 05
-0.42807741D 06	0.19906811D 06	-0.23980228D 07	-0.22173700D 03	0.12533913D 06	0.97202460D 05
-0.12798030D 05	-0.14573470D 02	0.37859157D 03	0.13917529D 03	-0.43200020D 03	-0.15724476D 01
-0.62765314D 06	0.19180333D 06	-0.38598014D 07	-0.49145041D 03	0.59278534D 06	0.83471247D 07
-0.17097219D 06	0.19180333D 06	-0.38598014D 07	-0.49145041D 03	0.59278534D 06	0.83471247D 07
-0.52475815D 06	0.18713070D 06	-0.54591321D 07	-0.11324958D 04	0.13384678D 07	0.11511948D 07
-0.14141990D 05	-0.14573470D 02	0.37859157D 03	0.13917529D 03	-0.43200020D 03	-0.15724476D 01
-0.70093562D 06	0.20043566D 05	-0.61215364D 06	0.15100514D 05	0.14787404D 07	0.44824900D 07
-0.19717483D 05	-0.11477171D 07	-0.12628005D 06	0.14559779D 02	0.64486937D 03	0.44824900D 07
-0.63094307D 06	0.19180333D 06	-0.38598014D 07	-0.49145041D 03	0.59278534D 06	0.83471247D 07
-0.15477730D 05	-0.14573470D 02	0.37859157D 03	0.13917529D 03	-0.43200020D 03	-0.15724476D 01
-0.77757816D 06	0.22752130D 05	-0.72606119D 06	0.16445566D 05	0.16355864D 07	0.46578227D 07
-0.21528242D 05	-0.23815158D 07	-0.16505884D 06	0.14568095D 02	0.64800570D 03	0.38544747D 07
-0.74651967D 06	0.18487566D 06	-0.63650269D 07	-0.16994685D 04	0.15976564D 03	-0.45023713D 03
-0.16807347D 05	-0.14573470D 02	0.37859157D 03	0.13917529D 03	-0.43200020D 03	-0.15724476D 01
-0.85785775D 06	0.45655480D 05	-0.84524540D 06	0.17784288D 05	0.18001770D 07	0.52330992D 07
-0.23335527D 05	-0.24951237D 07	-0.16565271D 06	0.14574715D 02	0.64739355D 03	0.35735364D 05
-0.87136647D 06	0.18405421D 06	-0.70211564D 07	-0.20248030D 04	0.16548610D 07	0.146607195D 08
-0.18129350D 05	-0.14573470D 02	0.37859157D 03	0.13917529D 03	-0.43200020D 03	-0.15724476D 01
-0.94204907D 06	0.20571860D 05	-0.56159973D 06	0.19116037D 03	0.19727639D 07	0.56081534D 07
-0.25143049D 05	-0.15029376D 07	-0.17241922D 06	0.14580060D 02	0.64657473D 03	0.19907460D 08
-0.10053506D 07	0.18320420D 06	-0.75325960D 07	-0.23774927D 04	0.21301408D 07	0.33307309D 05
-0.14443180D 05	-0.14573470D 02	0.37859157D 03	0.13917529D 03	-0.43200020D 03	-0.15724476D 01
-0.10304234D 07	0.28175682D 07	-0.11228770D 07	0.20446298D 05	0.72062300D 08	0.159007460D 08
-0.26647707D 05	-0.28739750D 07	-0.10645080D 06	0.14584564D 02	0.65374340D 03	0.46085650D 08
-0.11485303D 05	-0.12625635D 06	-0.32323232D 07	-0.27575606D 04	0.46296517D 07	0.62013547D 05
-0.20748199D 05	-0.14615114D 02	0.37176307D 04	0.34666265D 03	-0.38004992D 06	0.62013547D 05
-0.71031640D 08	0.26500771D 05	0.55891452D 07	0.44007940D 06	0.14306380D 05	0.26193604D 08
-0.28480290D 08	-0.12947462D 06	-0.17280299D 06	0.14995357D 01	0.58791913D 02	0.71205127D 04
-0.58626890D 07	-0.17624570D 06	-0.38011780D 06	-0.65757280D 05	-0.11311781D 08	0.76624207D 08
-0.40270011D 06	-0.15140600D 01	0.86291468D 03	0.15359694D 04	0.37495242D 04	-0.13843630D 04
-0.72032038D 08	0.24315785D 05	0.57494730D 07	0.44663065D 06	0.14506254D 08	0.26569560D 08
-0.28895433D 05	-0.13185740D 06	-0.17276839D 06	0.14994960D 01	0.58770404D 02	0.71205127D 04
-0.57225990D 07	0.17624570D 06	-0.38011780D 06	-0.65757280D 05	-0.11311781D 08	0.76624207D 08
-0.40635722D 06	-0.15140600D 01	0.86291468D 03	0.15359694D 04	0.37495242D 04	-0.13843630D 04
-0.73030825D 08	0.24315785D 05	0.57494730D 07	0.44663065D 06	0.14506254D 08	0.26569560D 08
-0.29310449D 05	-0.13185740D 06	-0.17276839D 06	0.14994960D 01	0.58770404D 02	0.71205127D 04
-0.58862370D 07	0.17624570D 06	-0.38011780D 06	-0.65757280D 05	-0.11311781D 08	0.76624207D 08
-0.41399150D 06	-0.15140600D 01	0.86291468D 03	0.15359694D 04	0.37495242D 04	-0.13843630D 04
-0.74026029D 08	0.24315785D 05	0.57494730D 07	0.44663065D 06	0.14506254D 08	0.26569560D 08
-0.29725297D 05	-0.13561532D 06	-0.17267134D 06	0.14995357D 01	0.58791913D 02	0.71205127D 04
0.60490420D 07	0.17624570D 06	-0.38011780D 06	-0.65757280D 05	-0.11311781D 08	0.76624207D 08
-0.46322920D 06	-0.15140600D 01	0.86291468D 03	0.15359694D 04	0.37495242D 04	-0.13843630D 04
-0.75023430D 08	0.24315785D 05	0.57494730D 07	0.44663065D 06	0.14506254D 08	0.26569560D 08
-0.30137976D 05	-0.13749294D 06	-0.17264060D 06	0.14995357D 01	0.58791913D 02	0.71205127D 04
-0.62153481D 07	0.17624570D 06	-0.38011780D 06	-0.65757280D 05	-0.11311781D 08	0.76624207D 08
-0.45244850D 05	-0.15140600D 01	0.86291468D 03	0.15359694D 04	0.37495242D 04	-0.13843630D 04
-0.76017607D 06	0.20974154D 05	0.64121079D 07	0.42996760D 06	0.15302763D 09	0.28071883D 08
-0.30554484D 05	-0.13937179D 06	-0.17257134D 06	0.14995357D 01	0.58791913D 02	0.71205127D 04
-0.93837300D 07	0.17624570D 06	-0.38011780D 06	-0.65757280D 05	-0.11311781D 08	0.76624207D 08
-0.43058717D 06	-0.15139848D 01	0.76195160D 05	0.16470037D 04	-0.37157916D 04	-0.34637596D 01
-0.77009940D 08	0.21368120D 05	0.65825391D 07	0.43457294D 06	0.15501063D 05	0.28447078D 08
-0.30968819D 05	-0.14124438D 08	-0.17252401D 06	0.14995357D 01	0.58791913D 02	0.71205127D 04
0.65541789D 07	0.17624570D 06	-0.38011780D 06	-0.65757280D 05	-0.11311781D 08	0.76624207D 08
-0.43645790D 06	-0.15139848D 01	0.76195160D 05	0.16		

Example Problem 2

The second example problem, a little more geometrically involved than the first problem, is the cap-torus-cylinder configuration shown in Fig. 11. To generate the required geometry, the shell is considered to be the combination of a spherical cap, a torus, and a cylinder. Hence, three segments are used for inputting the shell geometry. The profiles of the first two segments are circular and obviously, the profile of the cylinder is linear. The elements are concentrated in the area of the apex of the shell, close to the cap-torus intersection, around the torus, and near the clamped end of the cylinder with the maximum number of elements (50) being utilized. Obviously, for this case the advantage of using the segmentized procedure for inputting the geometry is quite evident since only nine cards are required to input the geometry for this complex configuration.

Since the thickness is constant for the shell, only one card is necessary for this data. The same is true for the mass and elastic properties. Again, a set of input data (Fig. 12) is presented and a portion of the output is included (Fig. 13).



SHELL DESCRIPTION

$$R_1 = 3.00 \text{ in.}$$

$$R_2 = 5.44 \text{ in.}$$

$$z_1 = 0.0755 \text{ in.}$$

$$z_2 = 2.5495 \text{ in.}$$

$$z_3 = 3.00 \text{ in.}$$

$$\beta = 2.88 \text{ deg.}$$

$$t = 0.125 \text{ in.}$$

MATERIAL PROPERTIES
ALUMINUM

$$E = 10.0 \times 10^6 \text{ psi}$$

$$G = 3.75 \times 10^6 \text{ psi}$$

$$\nu = 0.333$$

$$\rho = 0.0942 \text{ lb/in}^3$$

ELEMENT BREAKDOWN

Segment 1 - Sub-segment 1	4 elements
Sub-segment 2	8 elements
Sub-segment 3	8 elements
Segment 2 - Sub-segment 1	18 elements
Segment 3 - Sub-segment 1	4 elements
Sub-segment 2	4 elements
Sub-segment 3	4 elements

Fig. 11 CAP-TORUS - CYLINDER CONFIGURATION

```

NCASE=1

                                PRINTOUT OF INPUT DATA

10      20      30      40      50      60      70      80
12345678901234567890123456789012345678901234567890

5      4
*****
EXAMPLE PROBLEM NO. 2
*****
                                SAMMSOR USER'S MANUAL
                                CAP-TORUS-CYLINDER CONFIGURATION
                                ALL ELEMENTS HAVE THE SAME MATERIAL PROPERTIES AND THICKNESS
*****
50      6      1      50      10000000.      10000000.      3750000.      0.333      0.0942
1      1      50      0.0755      0.0      0.0755      3.00      87.12
1      1      50      0.125      0.0755      0.2      8      0.6      0.0
1      1      50      0.0755      0.0755      3.00      87.12
21      2      4      0.0755      0.0755      3.00      87.12
39      2      18      2.625      1.0      0      0.0      0.0      0.0
2      18      2.625      1.0      0      0.0      0.0      0.0
39      2.625      1.0      0      0.0      0.0      0.0
51      5.625      5.4400      0.00
1      4      0.2      0.6      0.2      0      0.0      0.0
END OF CASE

```

Fig. 12 INPUT DATA - EXAMPLE PROBLEM 2

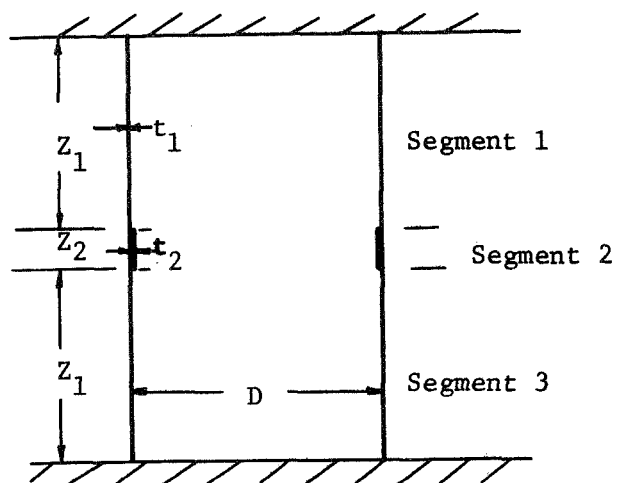
GEOMETRIC AND ELASTIC PROPERTIES OF STRUCTURE												
ELEM. NO.	NODE NOS.	Z	COORDINATES	R	SLOPE	S	THICKNESS	E1	ELASTIC CONSTANTS E2	G	NU1	RHO
1	1	0.0	0.100000-05	0.900000 02		0.75030-01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	2	0.188570-03	0.150060 00	0.898560 02								
2	2	0.188570-03	0.150060 00	0.898560 02		0.22510 00	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	3	0.754300-03	0.300130 00	0.897120 02								
3	3	0.754300-03	0.300130 00	0.897120 02		0.37520 00	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	4	0.169720-02	0.450190 00	0.895680 02								
4	4	0.169720-02	0.450190 00	0.895680 02		0.52520 00	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	5	0.301720-02	0.600240 00	0.894240 02								
5	5	0.301720-02	0.600240 00	0.894240 02		0.71280 00	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	6	0.570430-02	0.825320 00	0.892080 02								
6	6	0.570430-02	0.825320 00	0.892080 02		0.93790 00	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	7	0.923990-02	0.105040 01	0.889920 02								
7	7	0.923990-02	0.105040 01	0.889920 02		0.11630 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	8	0.136240-01	0.127540 01	0.887760 02								
8	8	0.136240-01	0.127540 01	0.887760 02		0.13880 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	9	0.188570-01	0.150050 01	0.885600 02								
9	9	0.188570-01	0.150050 01	0.885600 02		0.16130 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	10	0.249370-01	0.172550 01	0.883440 02								
10	10	0.249370-01	0.172550 01	0.883440 02		0.18380 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	11	0.318660-01	0.195050 01	0.881280 02								
11	11	0.318660-01	0.195050 01	0.881280 02		0.20630 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	12	0.396440-01	0.217540 01	0.879120 02								
12	12	0.396440-01	0.217540 01	0.879120 02		0.22880 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	13	0.482690-01	0.240040 01	0.876960 02								
13	13	0.482690-01	0.240040 01	0.876960 02		0.24390 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	14	0.513320-01	0.247530 01	0.876240 02								
14	14	0.513320-01	0.247530 01	0.876240 02		0.25140 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	15	0.544900-01	0.255030 01	0.875520 02								
15	15	0.544900-01	0.255030 01	0.875520 02		0.25890 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	16	0.577420-01	0.262530 01	0.874800 02								
16	16	0.577420-01	0.262530 01	0.874800 02		0.26640 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	17	0.610880-01	0.270020 01	0.874080 02								
17	17	0.610880-01	0.270020 01	0.874080 02		0.27390 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	18	0.645280-01	0.277520 01	0.873360 02								
18	18	0.645280-01	0.277520 01	0.873360 02		0.28140 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	19	0.680630-01	0.285010 01	0.872640 02								
19	19	0.680630-01	0.285010 01	0.872640 02		0.28890 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	20	0.716910-01	0.292510 01	0.871920 02								
20	20	0.716910-01	0.292510 01	0.871920 02		0.29640 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	21	0.754140-01	0.300000 01	0.871200 02								
21	21	0.755000-01	0.300000 01	0.871200 02		0.31090 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	22	0.954730-01	0.321530 01	0.822800 02								
22	22	0.954730-01	0.321530 01	0.822800 02		0.33260 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	23	0.133540 00	0.342820 01	0.774400 02								
23	23	0.133540 00	0.342820 01	0.774400 02		0.35420 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	24	0.189430 00	0.363700 01	0.726000 02								
24	24	0.189430 00	0.363700 01	0.726000 02		0.37580 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	25	0.262750 00	0.384050 01	0.677600 02								
25	25	0.262750 00	0.384050 01	0.677600 02		0.39750 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	26	0.352970 00	0.403700 01	0.629200 02								
26	26	0.352970 00	0.403700 01	0.629200 02		0.41910 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	27	0.459450 00	0.422520 01	0.580800 02								
27	27	0.459450 00	0.422520 01	0.580800 02		0.44070 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	28	0.581420 00	0.440370 01	0.532400 02								
28	28	0.581420 00	0.440370 01	0.532400 02		0.46230 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	29	0.718030 00	0.457130 01	0.484000 02								
29	29	0.718030 00	0.457130 01	0.484000 02		0.48400 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	30	0.868290 00	0.472680 01	0.435600 02								
30	30	0.868290 00	0.472680 01	0.435600 02		0.50560 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	31	0.103110 01	0.486910 01	0.387200 02								
31	31	0.103110 01	0.486910 01	0.387200 02		0.52720 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	32	0.120540 01	0.499710 01	0.338800 02								
32	32	0.120540 01	0.499710 01	0.338800 02		0.54890 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	33	0.138990 01	0.511000 01	0.290400 02								
33	33	0.138990 01	0.511000 01	0.290400 02		0.57050 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	34	0.158320 01	0.520680 01	0.242000 02								
34	34	0.158320 01	0.520680 01	0.242000 02		0.59210 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	35	0.178400 01	0.528710 01	0.193600 02								
35	35	0.178400 01	0.528710 01	0.193600 02		0.61380 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	36	0.199080 01	0.535010 01	0.145200 02								
36	36	0.199080 01	0.535010 01	0.145200 02		0.63540 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	37	0.220220 01	0.539540 01	0.068000 01								
37	37	0.220220 01	0.539540 01	0.068000 01		0.65700 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	38	0.241670 01	0.542270 01	0.484000 01								
38	38	0.241670 01	0.542270 01	0.484000 01		0.67860 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	39	0.263280 01	0.543190 01	-0.222640-13								
39	39	0.263280 01	0.544000 01	0.0		0.69700 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	40	0.277500 01	0.544000 01	0.0								
40	40	0.277500 01	0.544000 01	0.0		0.71200 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	41	0.292500 01	0.544000 01	0.0								
41	41	0.292500 01	0.544000 01	0.0		0.72700 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	42	0.307500 01	0.544000 01	0.0								
42	42	0.307500 01	0.544000 01	0.0		0.74200 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	43	0.322500 01	0.544000 01	0.0								
43	43	0.322500 01	0.544000 01	0.0		0.77200 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	44	0.367500 01	0.544000 01	0.0								
44	44	0.367500 01	0.544000 01	0.0		0.81700 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.333	0.09420
	45	0.412500 01	0.544000 01	0.0								
45	45	0.412500 01	0.544000 01	0.0		0.86200 01	0.12500 00	0.1000 08	0.1000 08	0.3750 07	0.33	

Example Problem 3

For illustrative purposes, a cylindrical shell (Fig. 14) with a stiffening ring at its midpoint has been selected as the third example. The input data for this shell should serve to illustrate many of the features of the code. In particular, the procedure used for a variable thickness shell will be demonstrated along with the option for inputting the nodal point geometry at each individual node.

To demonstrate these input procedures the cylinder will be idealized using nine elements: four on each side of the ring, and one containing the ring. If this shell were to actually be analyzed, many more elements would obviously be necessary. The elements on each side of the ring are considered to be five inches in length (s-direction) and one-eighth inch thick while the element for the stiffener is three inches long and one-fourth inch thick.

Two sets of input data are presented in Figures 15 and 16. The first set inputs separately the thickness of each element and the geometry for each node. The second data set effectively employs the program capabilities and inputs the thickness and geometry for the three segments of the cylinder.



SHELL DESCRIPTION

$D = 10$ in.
 $z_1 = 20$ in.
 $z_2 = 3$ in.
 $t_1 = 0.125$ in.
 $t_2 = 0.250$ in.

MATERIAL PROPERTIES
STEEL

$E = 30.0 \times 10^6$ psi
 $G = 11.55 \times 10^6$ psi
 $\nu = 0.3$
 $\rho = 0.289$ lb/in.³

ELEMENT BREAKDOWN

Segment 1 - 4 elements
 Segment 2 - 1 element
 Segment 3 - 4 elements

Fig. 14 STIFFENED CYLINDRICAL SHELL

```

NCASE= 3

CARD
TYPE
PRINTOUT OF INPUT DATA
10 20 30 40 50 60 70 80
12345678901234567890123456789012345678901234567890

II - A
- B
- B
- B
- B
- B
*****
EXAMPLE PROBLEM NO. 3
THIS EXAMPLE DEMONSTRATES THE PROCEDURE USED TO SPECIFY VARIABLE
THICKNESSES AND ALSO SHOWS THE PROCEDURE FOR SPECIFYING THE GEOMETRY OF
EACH NODE OF THE IDEALIZED SHELL
*****
9 5 1
1 9 30000000. 30000000. 11550000. 0.3 0.289
1 4 0.125
5 5 0.250
6 9 0.125
1 1 0.0 5.0 0.0
2 1 5.0 1.0 0 0.0 0 0.0 0 0.0
1 1 5.0 0.0 0.0 0.0
2 10.0 5.0 0.0 0.0 0.0
1 1 10.0 1.0 0 0.0 0 0.0 0 0.0
3 1 10.0 5.0 0.0 0.0 0.0
3 1 15.0 1.0 0 0.0 0 0.0 0 0.0
1 1 15.0 5.0 0.0 0.0 0.0
4 1 15.0 1.0 0 0.0 0 0.0 0 0.0
4 1 20.0 5.0 0.0 0.0 0.0
5 1 20.0 1.0 0 0.0 0 0.0 0 0.0
1 1 20.0 5.0 0.0 0.0 0.0
6 1 23.0 1.0 0 0.0 0 0.0 0 0.0
1 1 23.0 5.0 0.0 0.0 0.0
6 1 28.0 1.0 0 0.0 0 0.0 0 0.0
7 1 28.0 5.0 0.0 0.0 0.0
1 1 33.0 1.0 0 0.0 0 0.0 0 0.0
8 1 33.0 5.0 0.0 0.0 0.0
1 1 38.0 1.0 0 0.0 0 0.0 0 0.0
9 1 38.0 5.0 0.0 0.0 0.0
1 1 43.0 1.0 0 0.0 0 0.0 0 0.0
10 1 43.0 5.0 0.0 0.0 0.0
1 1 1.0 0 0.0 0 0.0 0 0.0
END OF CASE
VII

```

Fig. 15 INPUT DATA (SET #1) EXAMPLE PROBLEM 3

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6. Pian, T. H. H., "Derivation of Element Stiffness Matrices," AIAA Journal, Vol. 2, No. 3, March 1964, pp. 576-577.
7. Novozhilov, V. V. Foundations of the Nonlinear Theory of Elasticity, Graylock Press, Rochester, N. Y., 1956.

APPENDIX

Appendix 1 Description of SAMMSOR II Subroutines

Subroutine	Description
MAIN	Controls the flow of the program by calling the other subroutines to both obtain the structural stiffness and mass matrices for the shell and prepare the input tape for use in the other compatible programs.
INPUT	Reads input data and program control parameters and prints a description of the idealized structure.
SLOCOR	Reads coordinates of shell segments and controls the calculation of the z, r, ϕ coordinates for the specified element breakdown.
FLAT	Calculates the z, r, ϕ nodal coordinates for a shell segment with a linear profile.
CIRCUL	Calculates the z, r, ϕ nodal coordinates for shell segment with a circular profile.
PARABO	Calculates the z, r, ϕ nodal coordinates for a shell segment with a parabolic profile.
FUS	Used by subroutine PARABO to calculate arc lengths for the elements of the segment.
ELEMCA	Calculates geometrical parameters i.e., $r, \phi, \phi', \phi'', \sin \phi, \cos \phi$, etc. for each element between the nodal points. In addition, this subroutine calculates the transformation matrix $[A]$ which relates the shell generalized coordinates, q_i , to the generalized coefficients, α_i , of the displacement functions.
CRETAL	Calculates the values of integrals in the meridional direction which are to be used in calculating the terms in the mass and stiffness matrices.

Appendix 1 Continued

Subroutines	Description
CREATL	Evaluates the element stiffness matrices based on generalized coefficients. The element stiffness matrices are condensed in this subroutine.
CREMAS	Controls the calculation of the element mass matrices according to harmonic number.
AZERO	Generates the element mass matrices based on generalized coefficients for the zero harmonic only.
NONAB	Generates the element mass matrices based on generalized coefficients for harmonics other than zero.
MMPLT3	Premultiplies the element mass and stiffness matrices based on generalized coefficients by $[A]^t$ and post-multiplies by $[A]$ to form the element stiffness and mass matrices $[k]$ and $[m]$
SIMP	Evaluates the integral of a given function using Simpson's integration.
ASEMBL	Assembles element stiffness and mass matrices to form structural stiffness and mass matrices, $[K]$ and $[M]$, for each harmonic.

Appendix 2 Glossary of Significant FORTRAN Variables in SAMSOR II

Variable	Subroutine where variable is defined or calculated	Description
AK	MMPLT3	Element stiffness matrix [k]
AKK	MMPLT3	Element mass matrix, [m]
AL	CRETAL	Integral of functions used in the [L] and [EMASS] matrices.
AM	ELEMCA	Matrix $[B]^{-1}$ which relates the generalized α coordinates to the generalized shell co- ordinates.
AR	ELEMCA	Matrix of the radial coordinates, r, of the NET stations between the nodes of an element.
ARCL	ELEMCA	Arc length of an element (s-direction).
AS	ELEMCA	Matrix of the meridional distances, s, of the NET stations between the nodes of an element.
CHECK	ELEMCA	Matrix [A], which relates the shell gen- eralized coordinates to the generalized coefficients of the displacement functions.
COMENT	INPUT	Alphanumeric data printed as output for problem identification.
COSINE	ELEMCA	Matrix whose elements are the cosine of ϕ for NET stations along the element.
E1	INPUT	Matrix of Young's Moduli in the meridional direction, E_s
E2	INPUT	Matrix of Young's Moduli in the circumfer- ential direction, E_θ
EMASS	CREMAS	Element mass matrix based on generalized coefficients.

Appendix 2 Continued

Variable	Subroutine where variable is defined or calculated	Description
F	CREATL	Element stiffness matrix based on generalized coefficients.
FNU1	INPUT	Poisson's Ratio, $\nu_{s\theta}$
FNU2	INPUT	Poisson's Ratio, $\nu_{\theta s}$
G	INPUT	Shear Modulus, G [for an isotropic material $G = E/2(1+\nu)$].
IA	INPUT	Total number of harmonics for which the mass and stiffness matrices are to be determined.
ICLASS	SLOCOR	Designates the type of profile for a given segment of the shell.
NCARDS	INPUT	Number of comment cards used for problem identification.
ND	MAIN	Logical unit number of the binary scratch tape on which the input data is stored.
NELEMS	INPUT	Total number of elements used to idealize the structure.
NELEMT	SLOCOR	Element number of the first element of a segment.
NEQ	INPUT	Number of equilibrium equations (degrees of freedom) per harmonic.

Appendix 2 Continued

Variable	Subroutine where variable is defined or calculated	Description
NET	INPUT	Number of Simpson's integration stations used over the meridional length of the shell elements. NET is set equal to 29 in the program.
NH	INPUT	Total number of harmonics for which the mass and stiffness matrices are to be determined.
NNODES	INPUT	Total number of nodes, equal to NELEMS + 1.
NS	MAIN	Logical unit number of a binary scratch tape.
NSIZE	INPUT	The number of terms in the structural stiffness or mass matrix (in vector form) for a particular harmonic.
NT	INPUT	Logical unit number of the tape on which stiffness and mass matrices will be stored.
PHI1	SLOCOR	Slope of the shell at the first node of a segment.
PHI2	SLOCOR	Slope of the shell at the last node of a segment.
PHIMAT	ELEMCA	Matrix $[\Psi]$ which relates the generalized shell coordinates for an element to the global or cylindrical coordinates of the structure.
PHP	ELEMCA	$d\phi/ds$ for NET stations along an element.
PHPP	ELEMCA	$d^2\phi/ds^2$ for NET stations along the element (constant).
PHPRIM	ELEMCA	$d\phi/ds$ at the middle of an element.

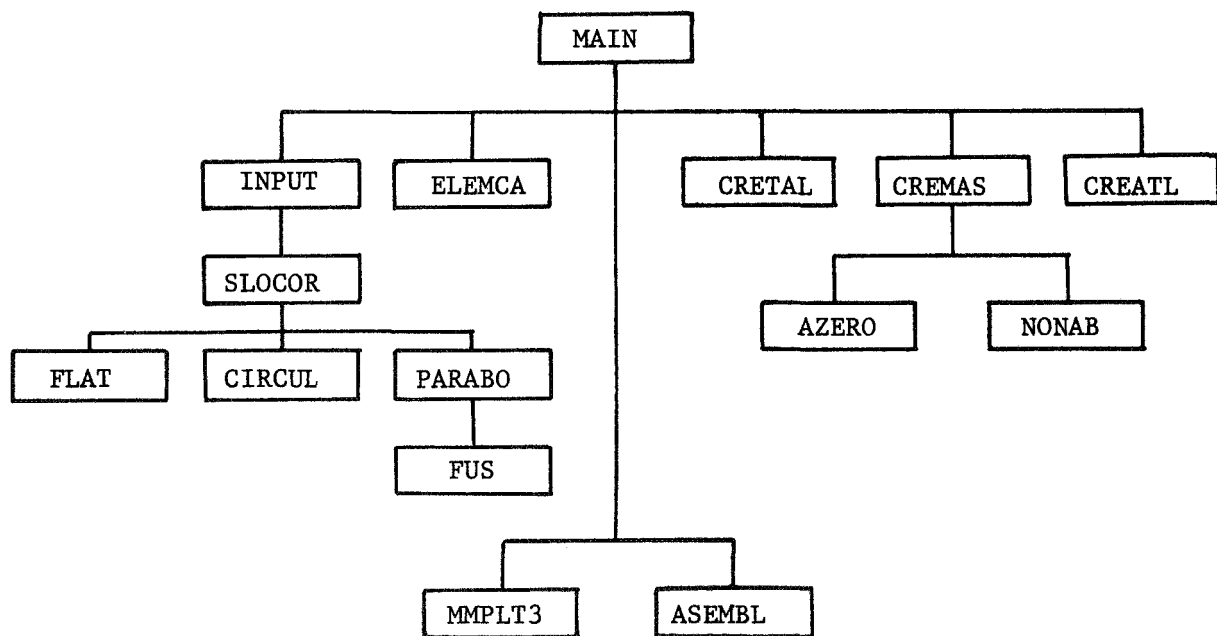
Appendix 2 Continued

Variable	Subroutine where variable is defined or calculated	Description
PHY1	INPUT	Slope at node i of element i
PHY2	INPUT	Slope at node $(i + 1)$ of element i
R1	SLOCOR	Radial coordinate of the first node of a segment.
R2	SLOCOR	Radial coordinate of the last node of a segment.
RA	SLOCOR	Decimal fraction corresponding to the lengths of the sub-segments into which a segment is being divided.
RHOS	INPUT	Matrix of the material densities of the elements, lbs/in ³
R01	INPUT	Radial coordinate of node i for element i
R02	INPUT	Radial coordinate of node $(i+1)$ for element i
SINE	ELEMCA	Matrix whose elements i are the sine of ϕ for the NET stations along the element.
STIFM	ASEMBL	Structural stiffness matrix, $[K]$, for a given harmonic.
THICK	INPUT	Matrix of element thicknesses.
TMASS	ASEMBL	Structural mass matrix, $[M]$, for a given harmonic.
Z1	INPUT	Axial coordiante of node i for element i
Z2	INPUT	Axial coordinate of node $(i+1)$ for ele- ment i
Z1C	SLOCOR	Axial coordinate of the first node of a segment.

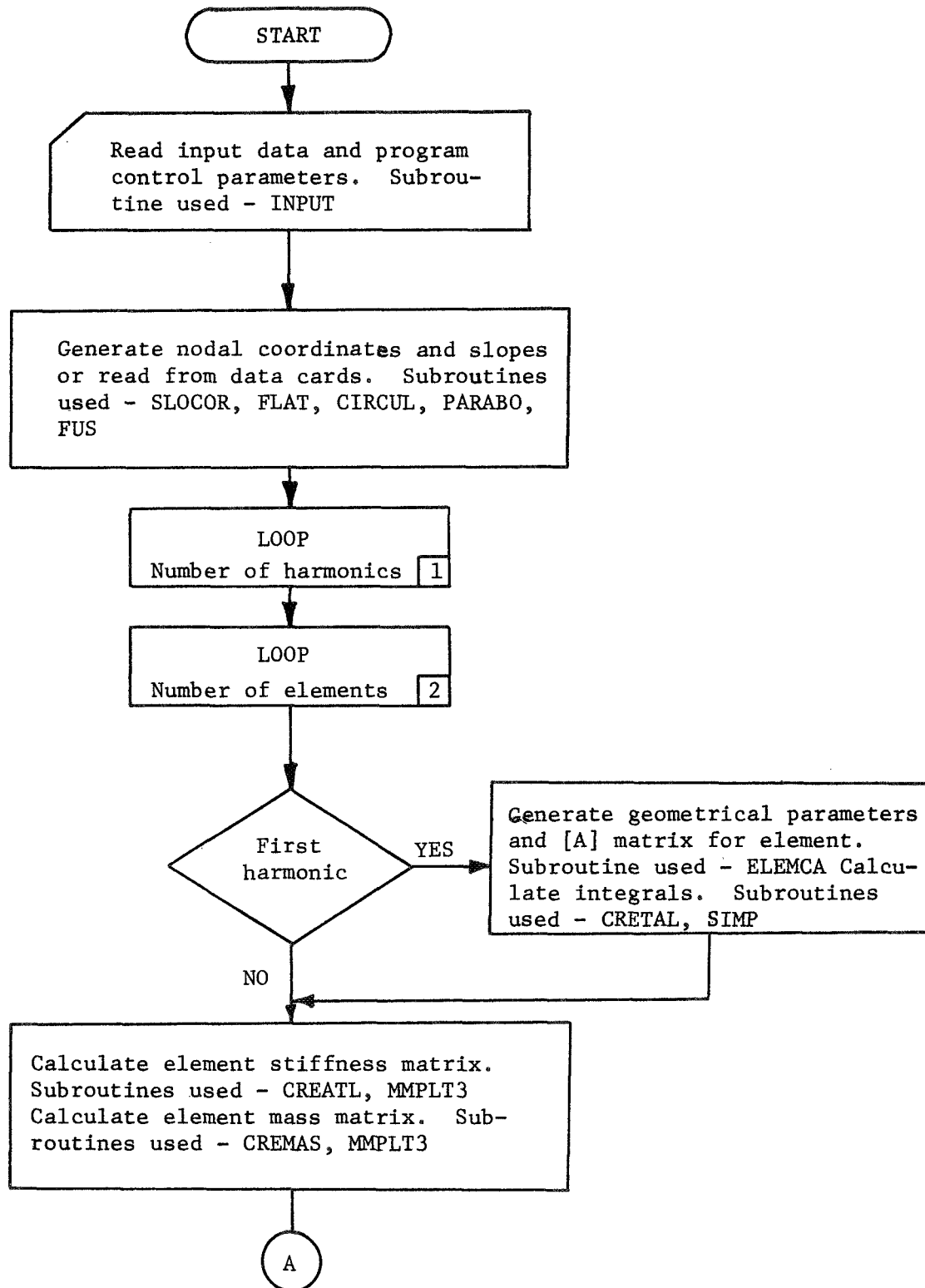
Appendix 2 Continued

Variable	Subroutine where variable is defined or calculated	Description
Z2C	SLOCOR	Axial coordinate of the last node of a segment.

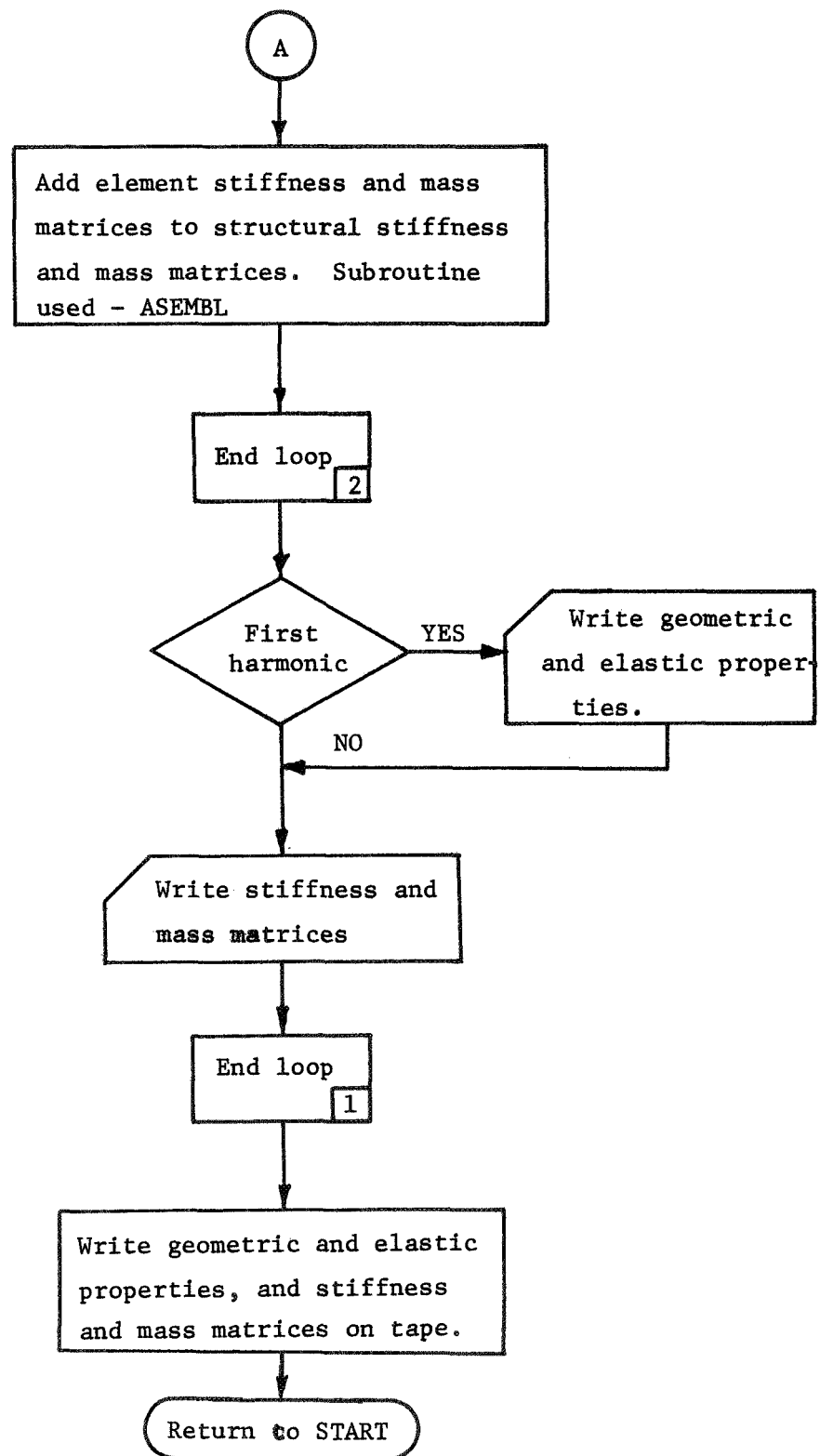
Appendix 3 Subroutine Call Map SAMSOR II



Appendix 4 Flow Chart of Basic Operations SAMMSOR II



Appendix 4 Continued



Appendix 5 Program Output

The output for the SAMMSOR code consists basically of four parts:

1. The input data for all cases comprising the run.
2. Alphanumeric data used for case identification.
3. Listing of the element properties and nodal point geometry.
4. The upper half of the diagonal band of the stiffness and mass matrices for each harmonic (if desired).

Section 2-4 are repeated for each of the cases of the run.

The output of part 1 is extremely helpful to the user since the input data for all cases is printed at the start of a run. If an error occurs in the program as a result of an error in the input data, then this printout should readily show the source of the error.

The comments of the user which describe or identify the case or problem considered are printed in the second section of the output. This information is stored on the output tape and is also printed as identifying data in the other SOR programs.

The fourth part of the output lists the lower half of the diagonal band of the structural stiffness and mass matrices. The program output and the vector form of storage used in the program contain only those terms within the band of the matrix which are on or above the diagonal. One line of output is printed for each degree of freedom when writing the stiffness and mass matrices. The last term on a particular line of output is the diagonal term corresponding to the degree of freedom. The terms which appear to the left of this diagonal term are the terms on the row (corresponding to the degree of freedom) which are within the

band of the matrix. In other words, the terms for the stiffness matrix of a particular harmonic with N degrees of freedom would appear as shown in Fig. A5-1. As is shown in Fig. A5-1, the maximum number of nonzero terms which appears in any row of the banded matrix is eight (8).

Terms whose value is zero will appear on the diagonals in the printout of the stiffness and mass matrices for harmonic zero. These terms correspond to the V-direction displacements which are identically zero for this harmonic. Since diagonal terms of zero value cause problems when solving a system of equations, these diagonal terms are set equal to one in the analysis programs to provide equations of the form

$$(1.0)q_n = 0.0$$

In addition to the four major parts of the printout, several checks have been included which generate output remarks if an error is encountered.

K_{11}							
K_{21}	K_{22}						
K_{31}	K_{32}	K_{33}					
K_{41}	K_{42}	K_{43}	K_{44}				
K_{51}	K_{52}	K_{53}	K_{54}	K_{55}			
K_{61}	K_{62}	K_{63}	K_{64}	K_{65}	K_{66}		
K_{71}	K_{72}	K_{73}	K_{74}	K_{75}	K_{76}	K_{77}	
K_{81}	K_{82}	K_{83}	K_{84}	K_{85}	K_{86}	K_{87}	K_{88}
K_{95}	K_{96}	K_{97}	K_{98}	K_{99}			
$K_{10,5}$	$K_{10,6}$	$K_{10,7}$	$K_{10,8}$	$K_{10,9}$	$K_{10,10}$		
-	-	-	-	-	-	-	
$K_{12,5}$	-	-	-	-	-	-	$K_{12,12}$
$K_{13,9}$	$K_{13,10}$	$K_{13,11}$	$K_{13,12}$	$K_{13,13}$			
-							
-							
-							
-							
-							
-							
$K_{N,N-7}$	$K_{N,N-6}$	-	-	-	-	-	$K_{N,N}$

Fig. A5-1 Banded Stiffness Matrix Output Format

Appendix 6 Modification of Program Capacity

The procedure for modifying the maximum number of elements which can be utilized in the SAMMSOR code is presented in this section. With only one exception, the modifications consist of changing only DIMENSION and COMMON statements. The following terms are defined to facilitate the modifications:

NE = Maximum number of elements

N1 = (26 * NE) + 10

N2 = 6 * NE

Having calculated these constants for the desired maximum number of elements, the following cards in the specified subroutines must be changed by substituting the values of the above constants.

Main Program

```
COMMON /B2/  R01(NE), R02(NE), Z1(NE), Z2(NE), PHY1(NE), PHY2(NE)
COMMON /B3/  E1(NE), E2(NE), FNU1(NE), FNU2(NE), G(NE), THICK(NE),
             RHOS(NE)
COMMON /B21/ F(8,8), AK(8,8), STIFM(N1)
COMMON /B30/ EMASS(8,8), AKK(8,8), TMASS(N1)
DIMENSION   XZ(N2), C0MENT(20), JUNK(20), CARD(20), SL(NE)
```

Subroutine INPUT

```
*COMMON /B2/
*COMMON /B3/
COMMON /B13/ SS(NE)
```

*Elements in this block are previously shown.

**NELMAX = NE

Subroutine SLØCØR

DIMENSION RA(5), NERAT(5), RØ1(NE), RØ2(NE), Z1(NE), Z2(NE),
PHII1(NE), PHII2(NE)

Subroutines FLAT, CIRCUL, and PARABØ

DIMENSION RØ1(NE), RØ2(NE), Z1(NE), Z2(NE), PHII1(NE), PHII2(NE)

Subroutine ELEMCA

*CØMMØN /B2/

*CØMMØN /B3/

*CØMMØN /B13/

Subroutine CRETAL

***CØMMØN /B3/

Subroutine AZERØ and NONAB

*CØMMØN /B3/

*CØMMØN /B30/

Subroutines MMPLT3 and ASEMBL

DIMENSION F(8,8), AK(8,8), STIFM(N1)

** This is the only card that must be changed which is not a CØMMØN or
DIMENSION statement.

***Replace G(NE) by GG(NE) in this block.

Appendix 7 Coding Sheet for Input Data

SHELL DESCRIPTION and comments	Status: Operational Deleted To be deleted on _____
	DATA SET NAME/NUMBER _____ DATE CREATED _____ CREATED BY _____
	SAMMSOR _____ VERSION _____ DOUBLE/SINGLE PREC. _____
CARD TYPE	INPUT DATA
II - A NCARDS NT (2I5)	
- B DESCRIPTIVE COMMENTS (20A4) _____ _____ _____ _____	
III NELEMS IA NPRNMS (3I5)	
IV IELM1 IELM2 EE1 EE2 GG FFNU1 RHO (2I5,5F10.0) _____ _____ _____ _____	
V IELM1 IELM2 THICKNESS (2I5, F10.0) _____ _____ _____ _____	

Appendix 7 Continued

VI SHELL GEOMETRY

- A NN1 Z R ϕ (I5, 3F10.0)

.....

- B NN2 Z R ϕ

.....

- C ICLASS NERAT RA NERAT RA NERAT RA NERAT RA NERAT RA
 1 1 2 2 3 3 4 4 5 5

.....

A
B
C

.....

A
B
C

.....

A
B
C

.....

A
B
C

.....

A
B
C

.....

A
B
C

.....

A
B
C

.....

VII END OF CASE

***** Add an END OF RUN card at the end of the final case

***** Add a Card I at the start of the first data set: This card contains NCASES,
ND, NS, and NS2 -- in that order