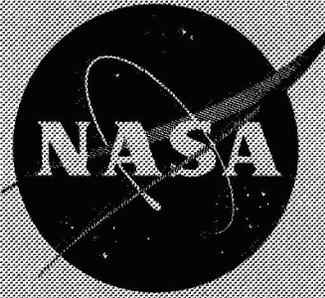


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MASTER AGREEMENT
TASK ORDER FIVE

Analyses and Limited Evaluation of
Payload and Legged Landing System Structures
For
The Survivable Soft Landing of Instrument Payloads

By O. R. Otto, R. M. Laurenson, R. A. Melliere, and R. L. Moore

Prepared by

MCDONNELL DOUGLAS ASTRONAUTICS COMPANY - EAST

St. Louis, Missouri 63166 (314) 232-0232

for Langley Research Center

NATIONAL AERONAUTICS AND SPACE ADMINISTRATION

WASHINGTON, D.C.

JULY 1971

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Prepared Under Contract No. NAS 1-8137 by

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ABSTRACT

This report describes two computer programs developed for the investigation of legged planetary landers. Analytical methods incorporated in the programs, operating instructions, and examples of program utilization are described. One program developed is the Structural Analysis Program, containing a Center Body Option, which can be used to analyze a structure by utilizing a finite element idealization. It also contains a Landing Gear Option for determination of the energy absorption and load-stroke characteristics of either a cantilever or inverted tripod landing gear configuration. The second program is the Landing Loads and Motions Program, used to predict spatial landing dynamics of a legged lander containing up to five landing gears. This program also contains options for including the effects of a flexible center body and determining lander stability.

Several exemplary computer runs are discussed to aid in interpretation of the operating instructions and to illustrate various available program options. Analytical results for several of these cases are compared to test data obtained during model test programs conducted at NASA Langley Research Center.

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1. INTRODUCTION

This report describes two computer programs developed by McDonnell Douglas Astronautics Company - East under NASA Contract NAS-1-8137(U) for investigation of legged type planetary landers. They are the Structural Analysis Program and the Landing Loads and Motions Program. The Structural Analysis Program contains options for the determination of internal center body load distributions, center body modal information, and the large displacement behavior of either an inverted tripod or a cantilever gear configuration. The Landing Loads and Motions Program can be employed for the determination of the spatial landing motions of a lander idealized either as a rigid body or as a flexible body. In addition, options are available for the determination of overall landing loads and lander stability boundaries.

Results of analyses conducted on typical legged lander configurations are presented and compared with test results obtained at NASA Langley Research Center. Test results were obtained using the Task Order Three Lander (Reference (1)) shown in Figure 1-1. These analyses demonstrate the capabilities of the developed computer programs.

The programs developed during Task Order Five provide the capability of conducting a complete design study on a legged lander beginning at the preliminary design phase and continuing through a final evaluation of the lander's structural configuration. A flow diagram of this capability is presented in Figure 1-2. Program options permit detailed analyses to be conducted at any point during the evolution of a legged lander. For example, analytical investigations for boiler plate models, scale models, or final lander configurations can be conducted.

During the preliminary design phase, various landing gear configurations are evaluated with the Landing Gear Option of the Structural Analysis Program. At the same time the landing loads and stability of a lander idealized with a rigid center body are investigated with the Landing Loads and Motions Program. Results of these analyses are used to select a final landing gear configuration. In addition, the predicted rigid body landing loads are used in conjunction with the Center Body Option of the Structural Analysis Program to determine internal load distributions in the center body structure. Based on

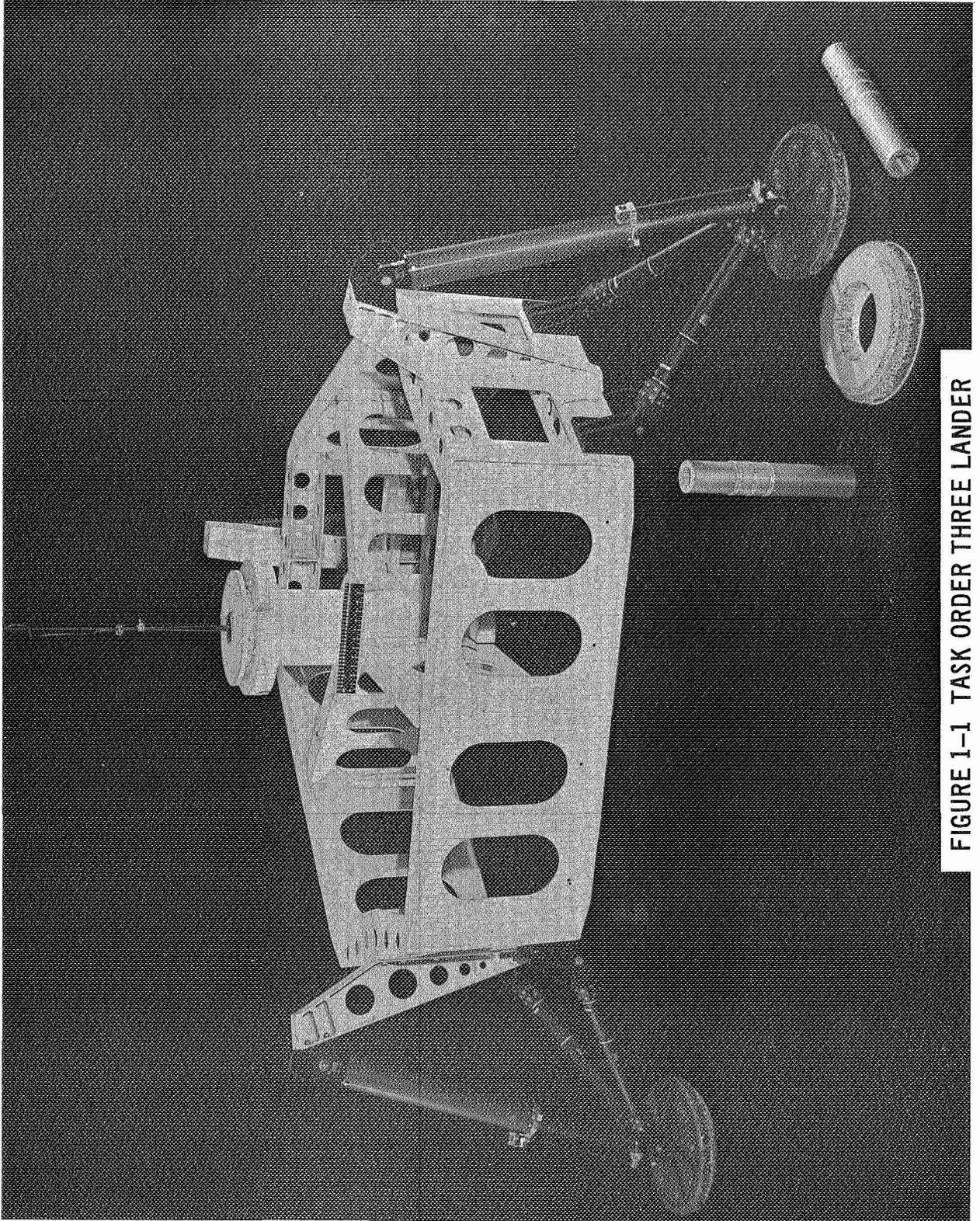


FIGURE I-1 TASK ORDER THREE LANDER

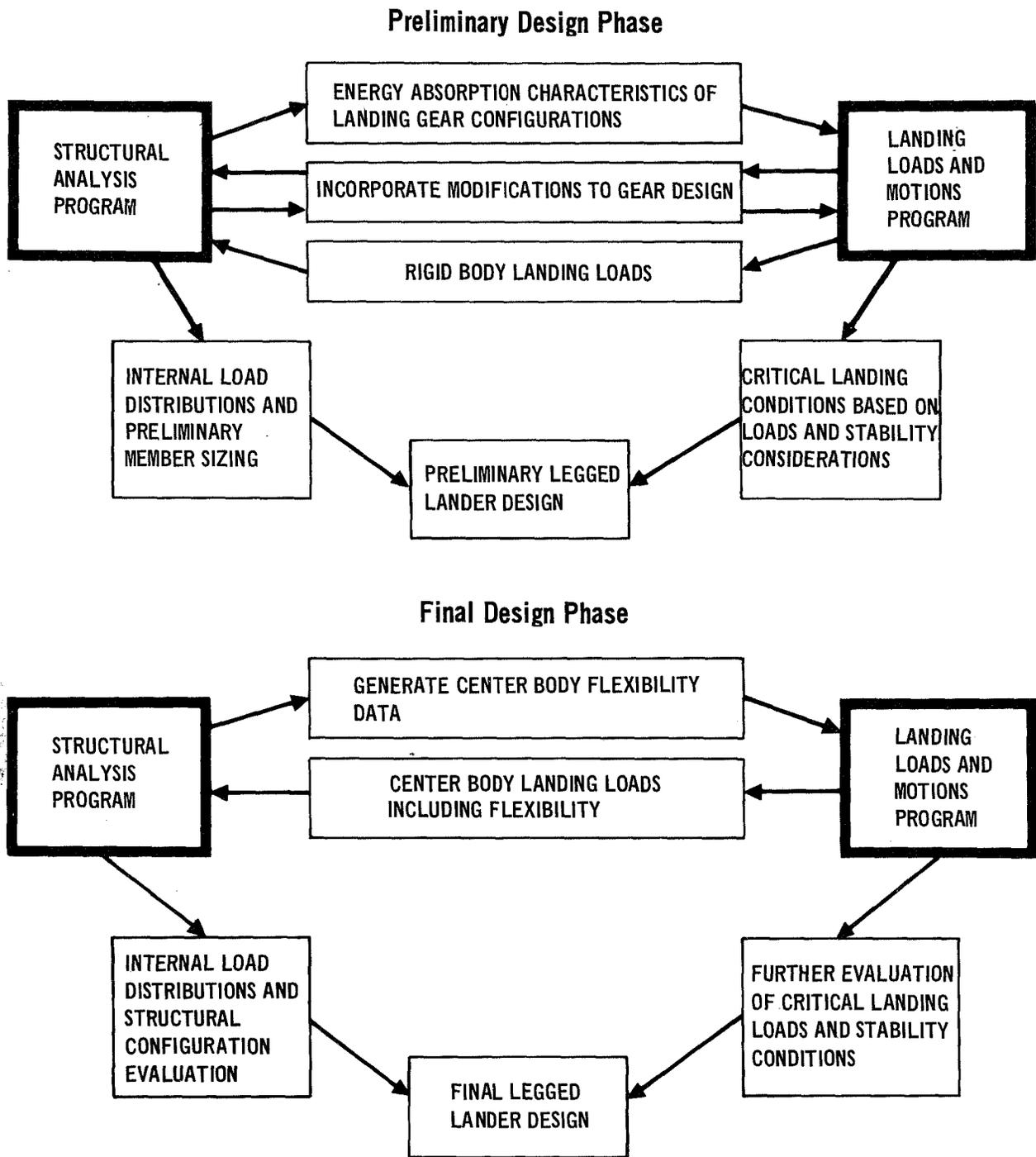


FIGURE 1-2 TASK ORDER FIVE PROGRAM USAGE DIAGRAM

the results of this preliminary design phase, the lander's structural arrangement is selected. At this time, the programs are used to determine center body flexibility information, landing loads, and stability conditions for the flexible lander, and internal load distributions for the final center body structural configuration.

2. COMPUTER PROGRAM CAPABILITIES

The two computer programs developed during Task Order Five contain many options which permit analyses of a wide variety of legged lander configurations. Legged lander configurations considered and the associated program capabilities and options are discussed in the following sections.

2.1 Legged Lander Description - The general arrangement of a typical legged lander configuration is shown in Figure 2-1. The legged lander is composed of three basic components: (1) center body, (2) payload and auxiliary equipment, and (3) landing gear. Items included in each component are shown in Figure 2-2.

The two landing gear configurations reflected in the analysis are shown in Figure 2-3. They are the inverted tripod gear and the cantilever gear. The main strut of both gears incorporates an energy absorption system. Drag struts may also include an energy absorption system or for the inverted tripod gear, they may simply be frame members stabilizing the main strut.

2.2 Structural Analysis Program - Two major options are available in the Structural Analysis Program. The first of these is the Center Body Option for the small displacement, finite element analysis of a legged lander center body structure. In formulating the Center Body Option, the Structural Analysis Program written for Task Order Two, Reference (2), was employed as a nucleus. During Task Order Five, many improvements and additions were made to this baseline program. In addition, a Landing Gear Option was developed for determining the large displacement stroking behavior of an individual landing gear. This is a major addition to the capabilities of the Task Order Two version of the Structural Analysis Program.

2.2.1 Center Body Option - The Center Body Option of the Structural Analysis Program is formulated for solution of redundant structures based on the small displacement finite element stiffness method and utilizing iterative methods of solution. The program can determine internal and external load distributions and deflection patterns for any system of forces or deflections impressed on a network of structural bar members and shear webs. Plastic

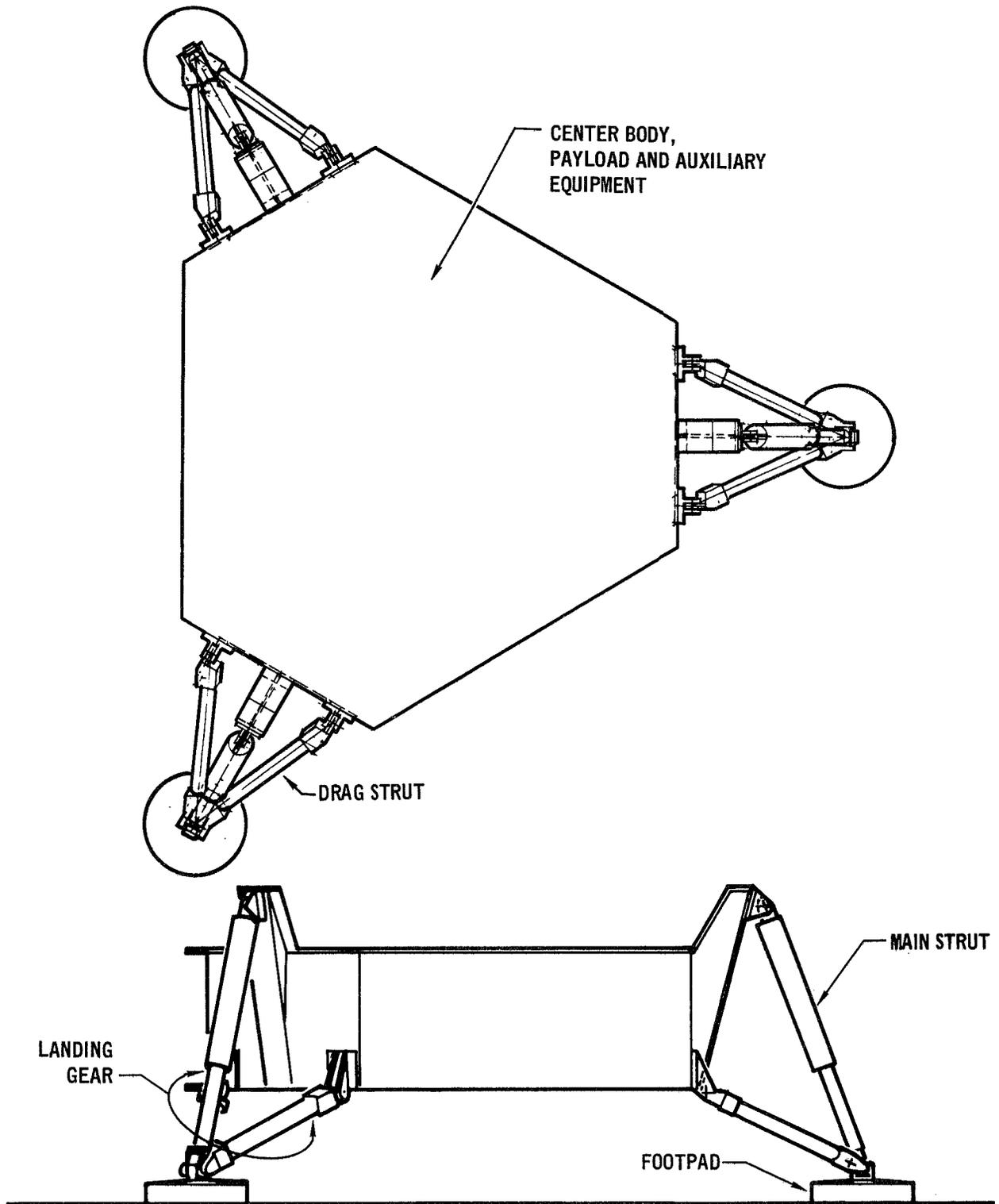


FIGURE 2-1 LEGGED LANDER

CENTER BODY
LANDING GEAR CARRY-THROUGH STRUCTURE
PAYLOAD SUPPORT STRUCTURE
AUXILIARY EQUIPMENT SUPPORT STRUCTURE

PAYLOAD AND AUXILIARY EQUIPMENT
FACSIMILE CAMERAS
ATMOSPHERIC SENSORS
SOIL ACQUISITION MECHANISM
SCIENCE INSTRUMENTS
COMMUNICATIONS EQUIPMENT
ELECTRONIC EQUIPMENT
POWER SUPPLY
TERMINAL PROPULSION SYSTEM
THERMAL CONTROL SYSTEM

LANDING GEAR
FOOTPAD
MAIN STRUT
DRAG STRUTS

FIGURE 2-2 LEGGED LANDER COMPONENTS

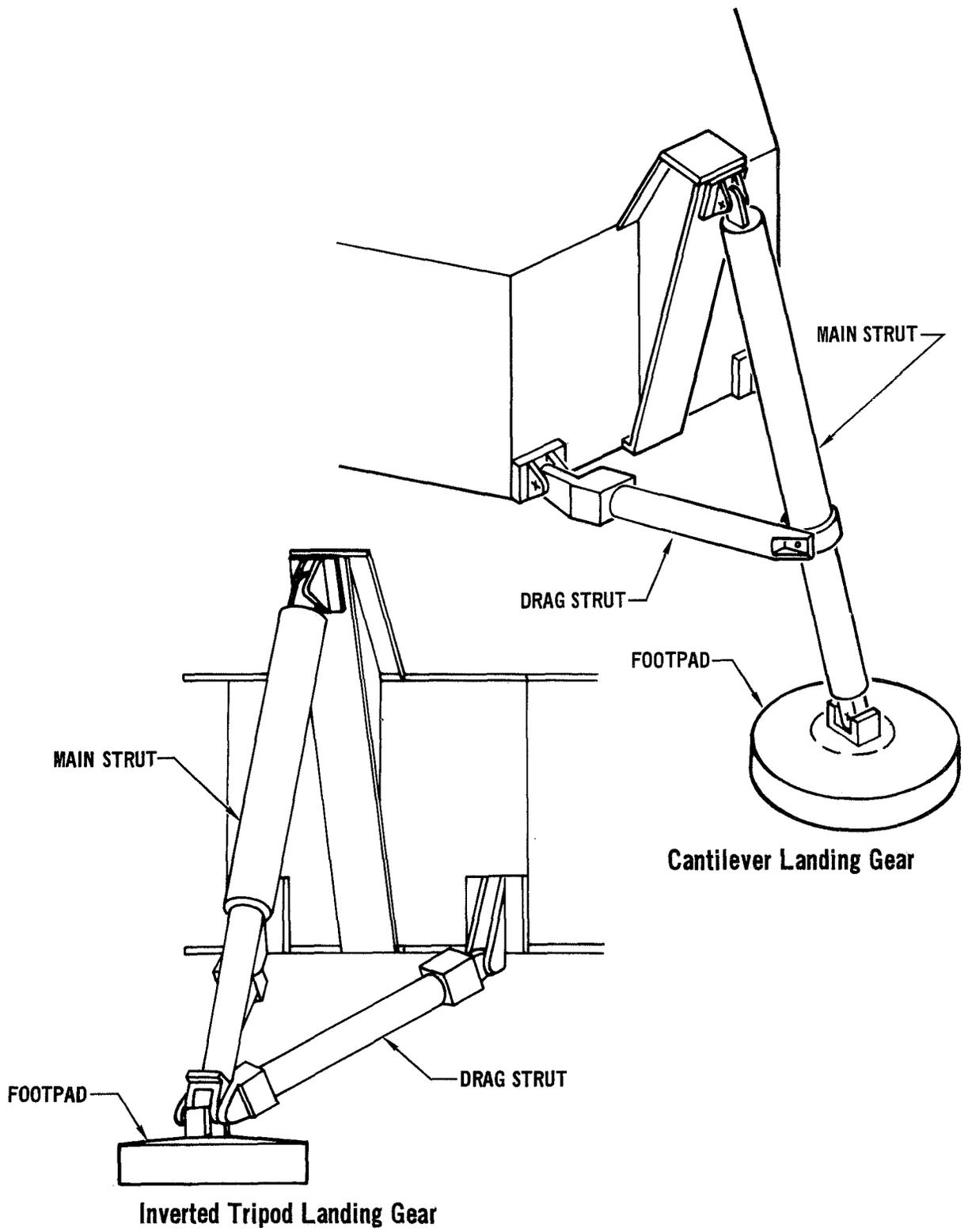


FIGURE 2-3 LANDING GEAR CONCEPTS

behavior of support structure can be simulated with the program. The program also allows simulation of cables, or any support members with restricted load carrying capability in certain directions. Mode shapes and natural frequencies may be obtained using this program. For modal analysis, a routine is incorporated to allow reduction in the size of the stiffness matrix for large structures, thus permitting a reduction in the required computer storage requirements and run time.

New capabilities available in this option of the Structural Analysis Program over the Task Order Two version follow. An optional iterative technique, the Conjugate Gradient Method, was added to increase program efficiency for solution of nodal deflections. A more efficient data input routine was programmed reducing core size required by this routine. The maximum number of bars which can be used to idealize a structure was increased from 74 to 130. A rectangular shear web element was added to expand idealization capability. An additional 2500 words of storage was provided for storing nonzero terms of the stiffness matrix. An improved matrix reduction routine was developed to increase efficiency when employing the modal analysis routine. The maximum number of lowest natural frequencies and associated mode shapes (excluding rigid body modes) that can be requested from the modal analysis routine was increased from 5 to 95. In addition, the capability for handling multiple data cases was added. These capabilities result in a program whose core storage requirements at execution utilize the maximum allowable storage (70K) specified for computer programs by NASA Langley Research Center.

The iterative method programmed for solution of nodal deflections was selected to minimize computer core storage allocated to the solution routine. It also permits minimization of computer time for a preliminary design problem, since accuracy requirements can be specified.

Both straight bar members and rectangular shear webs are available for idealization of the center body structure. As explained in Section 4.1.1.2, the program internally replaces a shear web by two diagonal bar elements. Hence the bar element is the basic element of the Center Body Option.

Each bar element is capable of carrying axial load, shear in two directions, bending in two directions, and torsion. Bar properties are

assumed to be constant between nodes (symmetry about one of the principal axes is assumed). The bar's longitudinal axis is assumed to pass through the centroidal axis and the bending neutral axes are assumed to align with centroidal axes. Torsional shear center of the bar is assumed to be on the bar's longitudinal axis.

Cables, which have one end attached to a support and the other to a structural joint, may be idealized by restricting the compression load carrying capability to zero, thus exercising the plastic analysis program logic.

Plastic behavior of support structure, such as crushable attenuators, may be simulated by inputting the force causing plastic deformation of each support member. An upper and lower limit must be input on the particular force or moment being constrained.

The program is based on small deflection theory with Hooke's Law applying, except with regard to the plastic support option of the program. Buckling of members is not considered, and coupling effects, such as occur with beam-columns, are neglected. Bars are assumed to be rigidly connected to each other, unless otherwise specified. Bars pinned at both ends may be simulated by setting the appropriate moments of inertia equal to zero. Bars pinned at one end and fixed at the other may be idealized through the use of special input indicators. Loads are applied to the joints as concentrated forces in global coordinates.

Computer core storage restrictions necessitated a number of limitations on the problem size which may be considered with the Center Body Option. As described in the operating instructions, this storage can be increased to accommodate larger problems. Within the current limitations, the allowable problem size is governed by the following considerations: up to 74 joints with six degrees of freedom at each joint, or a maximum assembled stiffness matrix 444 by 444; 130 bars maximum; 30 shear webs maximum, each of which reduces the allowable number of bars by two; 26 reference points maximum; 300 support constraints maximum; 88 plastic force constraints maximum (88 upper and 88 lower limits); and 88 nonzero input deflections maximum. Mode shapes and natural frequencies can be determined for a maximum assembled stiffness matrix size of 102 by 102 which corresponds to 17 joints with six

degrees of freedom or many joints with fewer degrees of freedom. A reduction routine permits modal analysis of large, complex structures because the associated large stiffness matrix (444 x 444 maximum) may be reduced to the allowable size (102 by 102) by selectively eliminating degrees of freedom. The program utilizes the mode shapes of the reduced system to generate mode shapes for all degrees of freedom of the original large stiffness matrix. The number of lowest natural frequencies and associated mode shapes requested from the modal analysis routine for any structure with free-free support must be a multiple of five and less than the order of the reduced stiffness matrix minus six.

2.2.2 Landing Gear Option - The Landing Gear Option of the Structural Analysis Program provides the capability for investigating the large displacement stroking behavior, energy absorption characteristics, and internal load distribution in an inverted tripod or cantilever landing gear. These characteristics are obtained by solving the displacement equations of equilibrium at specified steps of applied footpad joint parameters. These parameters can be specified in either of two ways: (1) a displacement normal to an arbitrarily oriented landing surface and the friction coefficient of the surface can be specified or (2) all three orthogonal displacement components in the Surface Coordinate System can be specified. The number of steps to be used for investigating the displaced state must be specified. The landing surface can be arbitrarily oriented with respect to the lander by specifying three Euler angles which orient the Surface Coordinate System relative to the Lander Coordinate System. A least squares solution technique is employed to solve the equations of equilibrium for the displacements at each step. The gear geometry and strut properties are arbitrary input quantities.

The program determines the external forces and displacements at the node points of the gear in the Surface Coordinate System, the internal loads in all struts, and the total energy components absorbed by the gear in both the Surface and Lander Coordinate Systems. The energy associated with crushing of footpad attenuation material can also be included for a footpad with up to three honeycomb crush levels.

The inverted tripod landing gear is idealized in the program with four nodes and three pin-ended elements representing the main strut and two drag struts. These elements have only axial load-carrying capability and may contain attenuation material with up to five crush levels in both tension and compression.

The cantilever landing gear is idealized in the program with five nodes and four elements. The main strut is idealized with two elements both of which have bending and axial load-carrying capability. These members are assumed to be rigidly connected at the center junction of the main strut. The junctions of the main strut with the center body and the footpad are assumed to be pinned. The drag struts are pin-ended with axial load-carrying capability only. The drag struts and the lower element of the main strut contain attenuation material with up to five crush levels in both tension and compression.

For both the inverted tripod and cantilever gear idealizations, all element properties are assumed constant between nodes. None of the elements of either gear is assumed to be capable of carrying torsional loads. The bending moment of inertia about any axis normal to the longitudinal axis of the main strut of the cantilever gear is assumed to be constant and beam-column effects are not included. Each strut of either gear may be of a different material.

2.3 Landing Loads and Motions Program - The Landing Loads and Motions Program predicts the landing dynamics of a legged lander. The lander is idealized as a center body structure to which the landing gears are attached. A small footpad which contacts the landing surface is located at the base of each landing gear. Program options are available for obtaining landing motions, landing loads, and stability information for planar or spatial landings on many different types of landing surfaces.

The lander center body may be idealized as either a rigid body or the effects of a flexible structure may be included. For a rigid center body up to six rigid body degrees of freedom may be included in the analysis. To conserve computer run time, any combination of center body rigid degrees of freedom may be suppressed when running planar landing cases. The flexible

center body is represented by the superposition of a number of free-free vibratory modes on the rigid body motion. From one to five modes may be included in the analysis. Flexible center body information may be obtained from either the Center Body Option of the Structural Analysis Program or some other eigenvalue program.

For a given legged lander configuration, up to five gears may be considered which may be either inverted tripod or cantilever gears. Each gear consists of a main strut and two drag struts which have pinned ends; thus, no moments or torques may be introduced at their ends. Both the main strut and drag struts are capable of carrying tension and compression loads and may possess velocity dependent force characteristics, elastic-plastic load-stroke characteristics, or a combination of the two. Five plastic load levels are available in both tension and compression for all of the landing gear struts. The load-stroke characteristics of all main struts in a given lander configuration are the same. Likewise, these characteristics for all the drag struts are the same, however, they may be different than the main struts. For a cantilever gear, the effect of main strut bending is included by modifying the elastic portion of the drag strut load-stroke relationship. Any combination of a constant magnitude Coulomb friction force, or a damping force dependent on the magnitude of the strut stroking velocity may be included in either the main struts, drag struts, or both. These friction forces are directed opposite to the strut stroking velocity. Relative motion between the center body and each footpad is employed in determining the magnitudes and directions of the landing gear strut loads.

Each footpad is represented as a single mass with three rigid body translational degrees of freedom. One degree of freedom is normal to the landing surface and the other two are in the plane of the landing surface. On an optional basis, a plastic load attenuation material, with up to three crush levels, may be located on the bottom of each footpad. For footpads whose equations of motion are not being integrated, the associated gears are assumed to be extensions of the center body structure and their inertia effects are included in the center body equations of motion.

Two soil mechanics routines are available for studying the footpad-soil interaction phenomenon. One method is similar to the footpad-soil interaction analysis developed during the Lunar Module Soil Mechanics Study. In this case, the soil is represented in terms of a number of semiempirical relationships. The second method determines the soil force through a simple elastic-plastic relationship between soil pressure and depth of soil penetration in conjunction with a coefficient of friction.

In addition to the standard program output defining the time histories of the footpad and center body motions and the load-stroke time histories in each landing gear strut, a number of additional output options are available. Accelerations at as many as ten points on the center body may be determined. This option allows the determination of landing accelerations for equipment items throughout the center body or correlation with test data obtained from accelerometers located at points other than the center of gravity.

The time history of the lander stability angle and pitching velocity of the lander in the direction of minimum stability may be obtained. This option, in conjunction with the spatial capabilities of the program and the two soil mechanics routines, allows a comprehensive study of the lander's stability characteristics to be made.

On an optional basis, all of the time history quantities required for defining the landing loads and acceleration patterns throughout the center body structure may be output on a magnetic tape. Points in time corresponding to possible high center body landing loads are determined by evaluating the printed output data obtained from the Landing Loads and Motions Program. The data on the magnetic tape may then be retrieved and the total landing load distribution throughout the center body structure determined at these specific time points of interest. The resulting loads are a combination of the inertia loads, gravity loads, and landing gear loads acting on the center body structure. These landing loads can then be input to the Center Body Option of the Structural Analysis Program and the internal loads in the individual structural members obtained.

Two numerical integration methods are incorporated in the program. These consist of a constant step Runge-Kutta method and a variable step Runge-Kutta

method. The constant step method allows definite user control over integration step size. The variable step method often results in less computer time for long runs such as required for stability cases.

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3. . STRUCTURAL DESIGN CRITERIA

In the design of a legged lander configuration, specific constraints and criteria are imposed on the design dependent on the mission to be accomplished by the vehicle. Presented in the following sections are typical design constraints and factors of safety to be used during the design process of a legged lander. These were used as guidelines in determining the capabilities and options to be incorporated in the two computer programs developed during Task Order Five. However, it is emphasized that these do not represent limitations on the computer programs.

3.1 Design Constraints - The following factors must be considered in landing system and payload structure design: simplicity, reliability, stowability, structural compatibility, environmental compatibility, weight and sterilizability. Methods must be provided for accomplishing postlanding payload exposure to permit operations of experiments such as bioscience and imagery; measurements of wind velocity and direction, ambient pressure, temperature, and humidity; determination of soil composition; and operation of systems such as power, communication, and thermal control. In addition, the following is a typical list of specific constraints which should be considered.

- (1) Mass of the landed vehicle shall be 584 kilograms.
- (2) The landed vehicle (center body structure and landing gear structure) shall be compatible with an 3.35 meter base diameter, 120° blunted cone entry vehicle.
- (3) Touchdown shall occur at a vertical velocity of 6.1 meters per second (relative to the gravity vector), and a maximum horizontal velocity of 3.7 meters per second.
- (4) The total mass of the scientific payload shall be a minimum of 175 kilograms and the payload packing density shall not exceed a maximum of 943 dynes/cm³.
- (5) The landing vehicle shall not be restricted in orientation about the roll axis.
- (6) Pitch and yaw attitudes at touchdown may vary as much as +10° from a plane normal to the gravity vector.

- (7) The landing vehicle shall have as a goal the capability of successfully landing on slopes of 30° to the local horizontal.
- (8) The landing system shall have the capability of performing satisfactorily when landing on surfaces containing particles varying in size from sand to 12.7 centimeter diameter rocks.
- (9) The atmospheric pressure at the surface shall be assumed to be nine mb.
- (10) The drag force on the footpad shall vary with penetration and applied normal forces.
- (11) The landing surface shall be assumed to have an average crushing stress of 41×10^8 dynes/m² for penetrations to depths of 15.2 centimeters, a constant density of 1414 dynes/cm³ for penetrations to depths of 15.2 centimeters and an angle of internal friction of 39 degrees.
- (12) The coefficient of sliding friction between the surface and the footpad shall be assumed to be 0.3.
- (13) Payload deceleration at any point in the payload shall be limited to a maximum of 20 earth g-units and landing deceleration of the footpads shall be limited to a maximum of 250 earth g-units.
- (14) Methods shall be available for accomplishing post landing payload exposure to permit operation of experiments such as bioscience and imagery; measurements of wind velocity and direction, ambient pressure, temperature, and humidity; determination of soil composition; and operation of systems such as power, communication, and thermal control.
- (15) Post landing orientation required by certain individual equipment items will be accomplished by aligning the item's positive X axis within +5 degrees to an axis perpendicular to the local surface slope.
- (16) Materials considered for use in the structures shall be compatible with space environment and a maximum temperature of 500°K and shall have minimum outgassing characteristics when exposed to a vacuum. Organic materials shall not be used in areas which may be subject to abrasion with the landing surface or to fragmentation with subsequent scattering of fragments on the landing surface.

(17) Surface gravitational acceleration shall be assumed to be 375 cm/sec^2 .

3.2 Factors of Safety - The following factors shall be applied to the maximum loads (limit loads) encountered in planetary landing within the constraints specified above.

Energy Absorbing Material for Landing Gear Struts	1.00
All Other Structures	1.25

The load obtained by multiplying limit load by the appropriate factor of safety is the ultimate load used in sizing the structure.

The landing gear system shall have capability for stroke greater than that required for landings within constraints defined herein. This additional stroke shall be available to provide clearance between the bottom of the center body and a rock lying on the surface.

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4. STRUCTURAL ANALYSIS PROGRAM

The Structural Analysis Program contains a Center Body Option and a Landing Gear Option. The Center Body Option determines internal load distributions in the center body structure, and generates modal data for use in the Landing Loads and Motions Program. Energy absorption capability of a single landing gear undergoing large displacement stroking motion is investigated with the Landing Gear Option.

In the Center Body Option, internal and external load distributions and deflection patterns are determined for any system of forces or deflections impressed on an elastic network of structural bar members and rectangular shear webs. Problems for which the support structure behaves plastically can be solved. In addition, mode shapes and natural frequencies can be obtained for a center body with free-free support.

The Landing Gear Option of the program is used to analyze the large displacement stroking behavior of a single inverted tripod or cantilever landing gear. Energy absorption characteristics of the gear and internal strut loads are obtained for any number of applied footpad displacement steps. The footpad can be required to stroke a given distance along any arbitrarily oriented straight line, or a given distance normal to an arbitrarily oriented frictional plane. In the latter case, the coefficient of friction between the footpad and the frictional plane determines the sliding path of the footpad in the plane.

Analytical methods employed in the linear small displacement and nonlinear large displacement finite element portions of the Structural Analysis Program are presented in Section 4.1. A discussion of the normal mode method for obtaining modal data is also included in this section. Organization of the program is described in Section 4.2 and information necessary to operate the program is presented in Section 4.3. A listing of the program is contained in Appendix H.

4.1 Analytical Methods - The landing system and payload structures are highly redundant space frames consisting of a network of members possessing extensional, flexural, and torsional stiffness. Several methods for solving complex structural problems are in use today that effectively utilize the

computer. Two methods considered for the Structural Analysis Program were the finite difference method and the finite element method. The finite element method was selected because of simplicity in dealing with nonhomogeneous, anisotropic structural applications. In addition, the elements can be changed easily in shape and size to follow complex boundary conditions or to allow for regions of rapid changes in stress or deflection.

A fundamental part of the finite element method is the technique used to obtain displacements at junctions of elements. Two approaches considered for the Structural Analysis Program were the force (flexibility) method and the stiffness (displacement) method. The stiffness method was selected primarily because the stiffness matrix developed may be used directly to generate the centerbody normal modes required to include centerbody flexibility in the Landing Loads and Motions Program. Also, the stiffness method lends itself readily to the large displacement technique necessary for landing gear analysis.

4.1.1 Center Body Option - The Center Body Option of the Structural Analysis Program can be used to analyze a structure idealized with an elastic network of structural bar elements and rectangular shear webs. The program internally replaces a rectangular shear web by two diagonal bar elements whose stiffness characteristics are equivalent to the shear web. Hence, the basic element of the program is the bar element.

4.1.1.1 Coordinate System - Two coordinate systems employed in the Center Body Option are the Local Coordinate System and the Global Coordinate System. Each bar member has its set of local coordinates as shown in Figure 4-1. Displacement notation of a general bar element capable of carrying axial load, shear in two directions, bending in two directions, and torsion is also indicated in Figure 4-1 for the Local Coordinate System. Subscript 1 refers to a displacement due to axial load, subscripts 2 and 3 are for displacement due to shear loads, subscript 4 is for a rotation due to torque, and subscripts 5 and 6 are for rotations due to moments. The Local Coordinate System origin for each bar is located at joint "p," with the X_ℓ axis aligned along the member axis. Positive X_ℓ is on the side of joint "p" towards joint "q." The Y_ℓ axis is perpendicular to X_ℓ and is located in the "pqr" plane. Positive Y_ℓ is on the side of X_ℓ towards point "r." The Z_ℓ

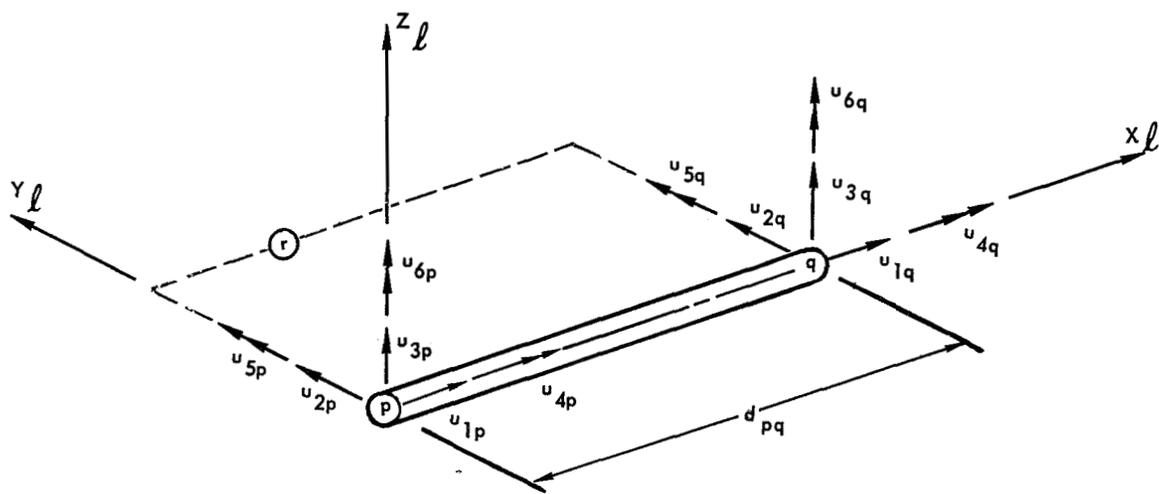


FIGURE 4-1 DISPLACEMENTS OF GENERAL BAR ELEMENT
IN LOCAL COORDINATE SYSTEM

axis is then established using the right hand rule. Local Coordinate Systems are established by identifying each bar's origin (joint "p"), end (joint "q"), and orientation of the bending axis Y_ℓ (defined by joint "r"). Moment of inertia I_T about the Y_ℓ axis, moment of inertia I_N about the Z_ℓ axis, cross-sectional area A , torsional constant J , modulus of elasticity E , and shear modulus G are specified for each bar relative to its Local Coordinate System.

A common coordinate system (global) for all structural elements must be established so that element forces and displacements may be related to a common frame of reference. This Global Coordinate System may be any convenient orthogonal right hand system. Displacement notation of the bar element in the Global Coordinate System is shown in Figure 4-2. Joint locations, external load distributions, and joint deflections are all specified in the Global Coordinate System.

4.1.1.2 Stiffness Matrices - A stiffness matrix for each bar element is generated in its Local Coordinate System based on small deflection theory. This is done by applying a unit displacement or rotation to one end of the bar (while restraining all other rotations and displacements) and determining the induced forces and moments. Displacements and rotations are applied sequentially until all degrees of freedom at each end of the bar element have been included. Displacement notation of a general bar element capable of carrying axial load, shear in two directions, bending in two directions, and torsion was shown in Figure 4-1.

Force-displacement relationships for the three unit displacements and three unit rotations possible at each end of the bar element are shown in Figure 4-3. Forces and displacements in the Local Coordinate System are related in matrix form by

$$\{F\} = [K] \{\delta_L\} \quad (4-1)$$

In Equation (4-1), $[K]$ is the matrix of stiffness coefficients; $\{F\}$ is a column matrix of applied forces at the joints, and $\{\delta_L\}$ is the column matrix of joint displacements. Application of Maxwell's Reciprocal Law allows formulation of a symmetric stiffness matrix when an orthogonal coordinate system is employed. The element stiffness matrix for a general bar is given in

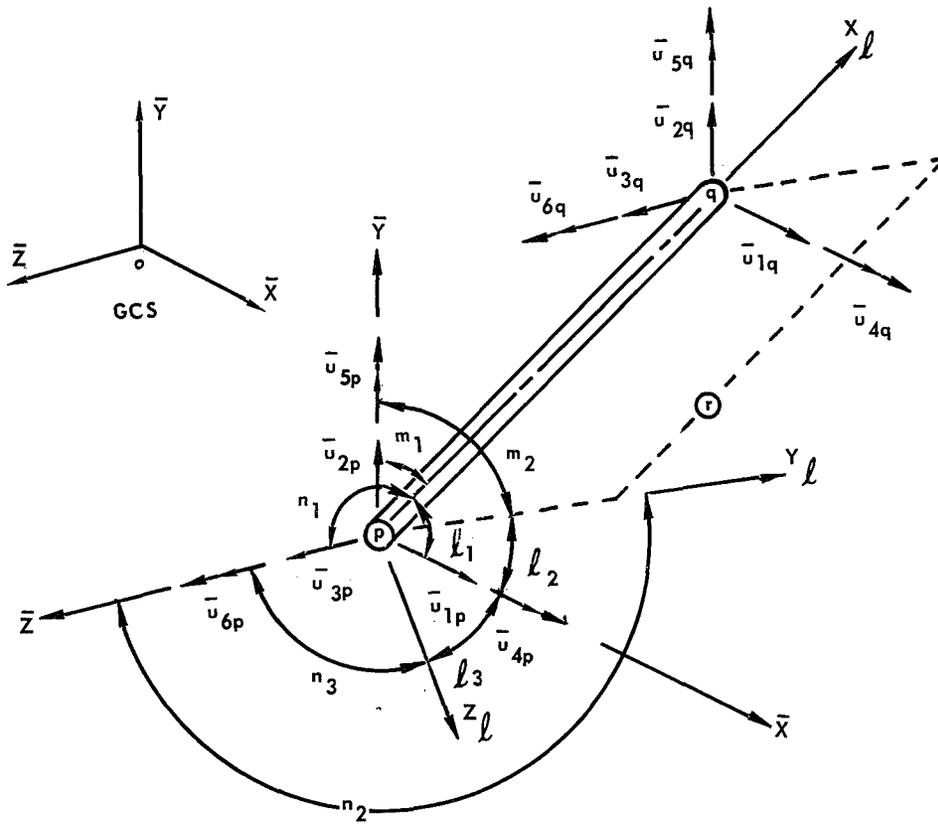


FIGURE 4-2 DISPLACEMENTS OF GENERAL BAR ELEMENT
IN GLOBAL COORDINATE SYSTEM

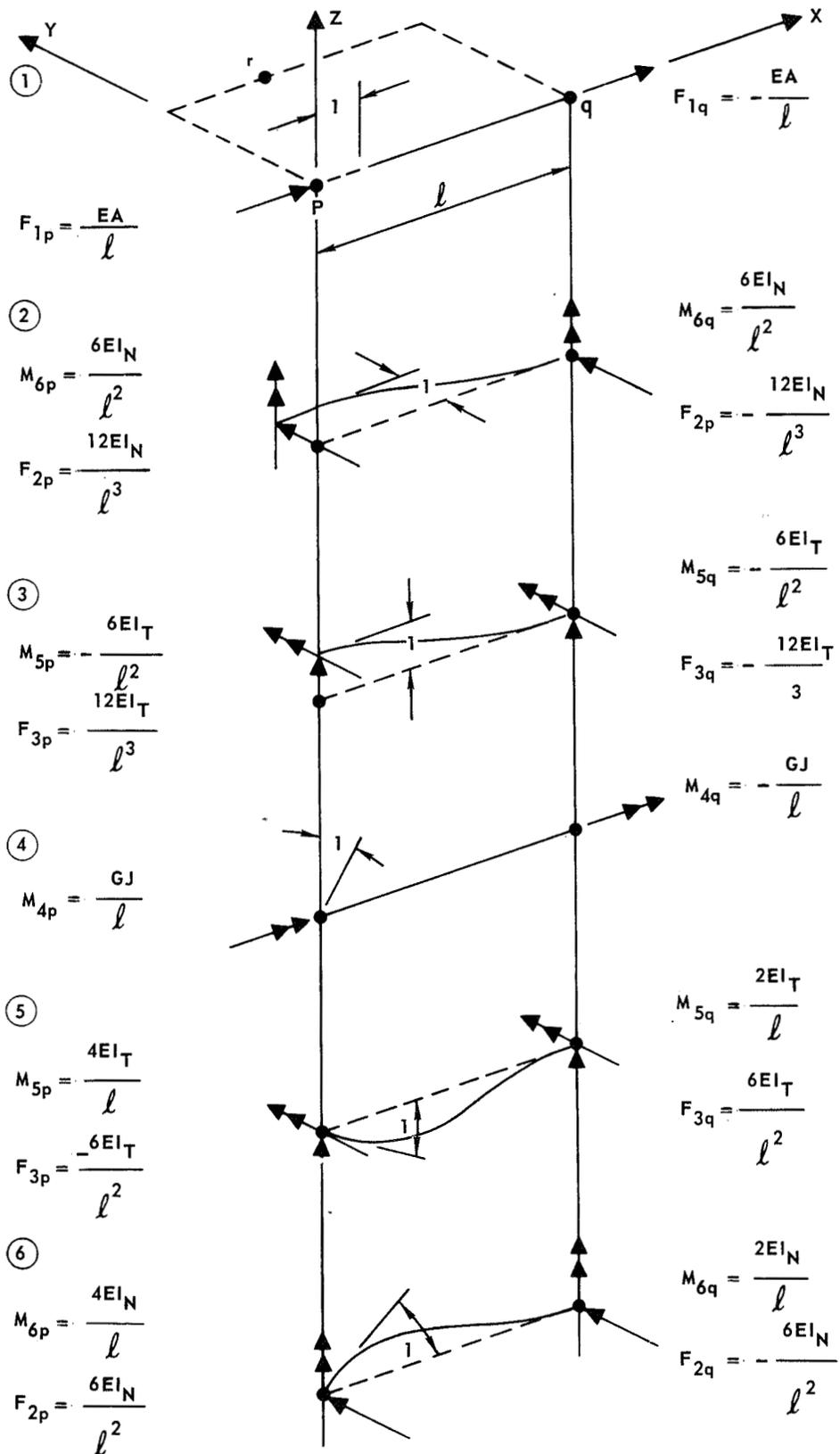


FIGURE 4-3 FORCE - DISPLACEMENT RELATIONS FOR GENERAL BAR ELEMENT

Figure 4-4. Terms in this matrix are the stiffness coefficients given in Figure 4-3.

If it is desired to include the effects of shearing strain on elemental beam deflection, the program applies the stiffness factors given in Figure 4-5 to the corresponding terms in Figure 4-4. Input shear form factors K_N and K_T are determined by dividing the total cross-sectional area of the bar by the area effective in carrying shear loads. The area used in determining K_N should be the area effective in carrying shear in the local Y_ℓ direction (bending about the Z_ℓ axis), while K_T is determined based on effective area for shear in the Z_ℓ direction. An example calculation of shear form factors K_N and K_T is given in Figure 4-6.

In addition, a rectangular shear web element is incorporated in the Center Body Option. The shear web idealization used by the program is indicated in Figure 4-7. Here the shear web is replaced by two diagonal bar elements whose stiffness characteristics are equivalent to the shear web. Cross-sectional properties of diagonals are derived by requiring equivalent strain energies in shear web and diagonals for statically equivalent internal loads.

Both ends of a member may be pinned (allowed to freely rotate) in a given direction by setting the appropriate moment of inertia equal to zero. If it is desired to pin one end of a member, while the other end remains fixed, the program utilizes the terms shown in Figures 4-8 through 4-11 in place of the corresponding terms in Figure 4-5. Terms used depend on the joint (p or q) pinned and the bending axis (Y_ℓ or Z_ℓ) about which the end is allowed to rotate. For example, if it is desired to pin the q end of a member with respect to bending about the Y_ℓ axis, terms in Figure 4-11 would be used in place of the corresponding terms in Figure 4-5.

To determine the stiffness matrix for the completely assembled structure, all element forces and displacements are related to the Global Coordinate System. Displacement notation of the bar element in Global Coordinate System is defined in Figure 4-2.

Transformation matrices are needed to change the frame of reference of each element from the Local to the Global Coordinate System. This transformation is expressed by the linear matrix equation

$$\{\delta_L\} = [\lambda]\{\delta_G\} \quad (4-2)$$

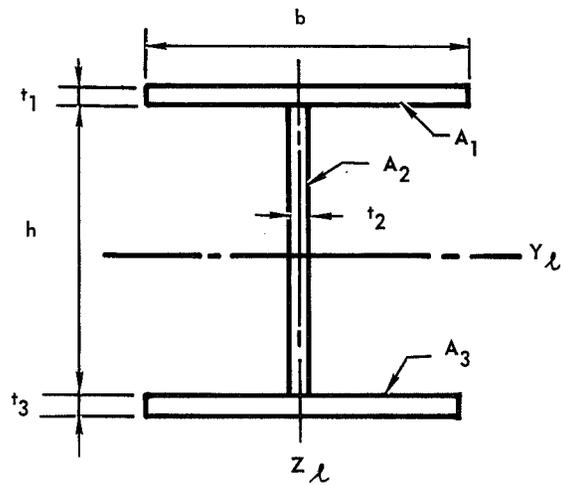
$$\mathbf{K} = \begin{matrix} & \begin{matrix} u_{1p} & u_{2p} & u_{3p} & u_{1q} & u_{2q} & u_{3q} & u_{4p} & u_{5p} & u_{6p} & u_{4q} & u_{5q} & u_{6q} \end{matrix} \\ \begin{matrix} F_{1p} \\ F_{2p} \\ F_{3p} \\ F_{1q} \\ F_{2q} \\ F_{3q} \\ M_{4p} \\ M_{5p} \\ M_{6p} \\ M_{4q} \\ M_{5q} \\ M_{6q} \end{matrix} & \begin{bmatrix} \frac{EA}{l} & & & -\frac{EA}{l} & & & & & & & & & \\ & \frac{12EI_N}{l^3} & & -\frac{12EI_N}{l^3} & & & & & \frac{6EI_N}{l^2} & & \frac{6EI_N}{l^2} & & \\ & & \frac{12EI_T}{l^3} & & -\frac{12EI_T}{l^3} & & -\frac{6EI_T}{l^2} & & & & -\frac{6EI_T}{l^2} & & \\ -\frac{EA}{l} & & & \frac{EA}{l} & & & & & & & & & \\ & -\frac{12EI_N}{l^3} & & \frac{12EI_N}{l^3} & & & & & \frac{6EI_N}{l^2} & & -\frac{6EI_N}{l^2} & & \\ & & -\frac{12EI_T}{l^3} & & \frac{12EI_T}{l^3} & & \frac{6EI_T}{l^2} & & & & \frac{6EI_T}{l^2} & & \\ & & & & & \frac{GJ}{l} & & & & -\frac{GJ}{l} & & & \\ & & \frac{6EI_T}{l^2} & & & \frac{6EI_T}{l^2} & \frac{4EI_T}{l} & & & & \frac{2EI_T}{l} & & \\ \frac{6EI_N}{l^2} & & & -\frac{6EI_N}{l^2} & & & & & \frac{4EI_N}{l} & & & \frac{2EI_N}{l} & \\ & & & & & -\frac{GJ}{l} & & & & \frac{GJ}{l} & & & \\ & & -\frac{6EI_T}{l^2} & & & \frac{6EI_T}{l^2} & \frac{2EI_T}{l} & & & & \frac{4EI_T}{l} & & \\ \frac{6EI_N}{l^2} & & & -\frac{6EI_N}{l^2} & & & & & \frac{2EI_N}{l} & & & \frac{4EI_N}{l} & \end{bmatrix} \end{matrix}$$

FIGURE 4-4 STIFFNESS MATRIX FOR GENERAL BAR ELEMENT

$$\begin{matrix}
 & u_{1p} & u_{2p} & u_{3p} & u_{1q} & u_{2q} & u_{3q} & u_{4p} & u_{5p} & u_{6p} & u_{4q} & u_{5q} & u_{6q} \\
 \begin{matrix}
 F_{1q} \\
 F_{2p} \\
 F_{3p} \\
 F_{1q} \\
 F_{2q} \\
 F_{3q} \\
 M_{4p} \\
 M_{5p} \\
 M_{6p} \\
 M_{4q} \\
 M_{5q} \\
 M_{6q}
 \end{matrix}
 &
 \left[\begin{array}{cccccccccccc}
 1 & & & & & & & & & & & & \\
 & C_{1N} & & & C_{1N} & & & & & C_{1N} & & & C_{1N} \\
 & & C_{1T} & & & C_{1T} & & & C_{1T} & & & C_{1T} & \\
 1 & & & 1 & & & & & & & & & \\
 & C_{1N} & & & C_{1N} & & & & & C_{1N} & & & C_{1N} \\
 & & C_{1T} & & & C_{1T} & & & C_{1T} & & & C_{1T} & \\
 & & & & & & & 1 & & & & 1 & \\
 & & C_{1T} & & & C_{1T} & & & C_{2T} & & & C_{3T} & \\
 & C_{1N} & & & C_{1N} & & & & & C_{2N} & & & C_{3N} \\
 & & & & & & & 1 & & & & 1 & \\
 & & & C_{1T} & & & C_{1T} & & & C_{3T} & & & C_{2T} \\
 & C_{1N} & & & C_{1N} & & & & & & C_{3N} & & C_{2N}
 \end{array} \right]
 \end{matrix}$$

$$K_S = \begin{matrix}
 C_{1N} = \frac{1}{1 + a_N} & C_{1T} = \frac{1}{1 + a_T} \\
 C_{2N} = 1 - \frac{3}{4(1 + 1/a_N)} & C_{2T} = 1 - \frac{3}{4(1 + 1/a_T)} \\
 C_{3N} = 1 - \frac{3}{2(1 + 1/a_N)} & C_{3T} = 1 - \frac{3}{2(1 + 1/a_T)} \\
 a_N = \frac{12EI_N/l^3}{GA/l} \cdot K_N & a_T = \frac{12EI_T/l^3}{GA/l} \cdot K_T
 \end{matrix}$$

FIGURE 4-5 STIFFNESS FACTORS APPLIED TO ORIGINAL MATRIX TO ACCOUNT FOR ELEMENTAL BAR SHEAR STRAIN



$$A_1 = bt_1$$

$$A_2 = ht_2$$

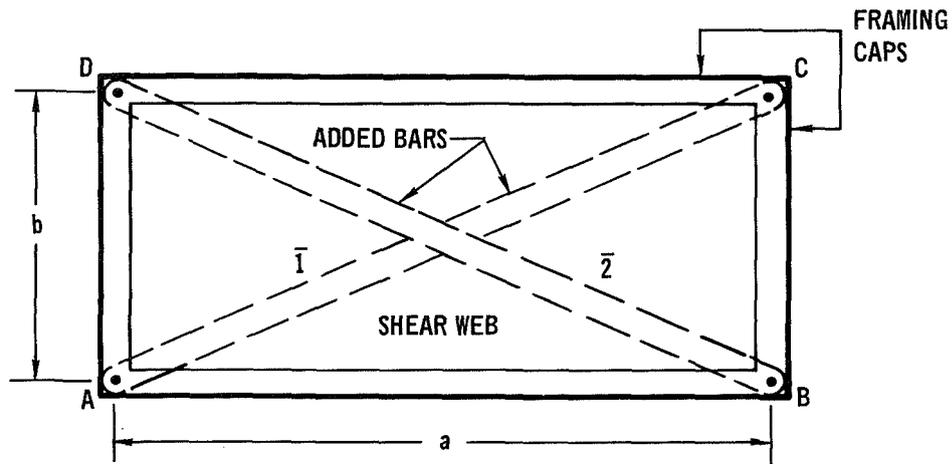
$$A_3 = bt_3$$

$$A_{TOTAL} = A_1 + A_2 + A_3$$

$$K_N = \frac{A_{TOTAL}}{A_1 + A_3}$$

$$K_T = \frac{A_{TOTAL}}{A_2}$$

FIGURE 4-6 EXAMPLE CALCULATION OF SHEAR FORM FACTOR



GIVEN SHEAR WEB:

MODULUS OF ELASTICITY E , POISSON'S RATIO ν , AND THICKNESS t .
 CORNER JOINTS A, B, C, D NUMBERED COUNTERCLOCKWISE.

DIAGONAL REPLACEMENT OF WEB BY PROGRAM:

ADD 2 BARS AS SHOWN. BARS ARE PINNED AT BOTH ENDS AND NOT ATTACHED TO EACH OTHER.
 BAR PROPERTIES ARE AS FOLLOWS:

$$\text{AREA} = t \cdot (a^2 + b^2)^{3/2} / 4ab (1 + \nu).$$

$$E = E_{\text{web}}$$

OTHER PROPERTIES = 0

WHERE a = LENGTH AB

b = LENGTH AD

INTERPRETATION OF PROGRAM OUTPUT:

$$\text{SHEAR FLOW IN WEB; } q = (P_{\bar{2}} - P_{\bar{1}}) / \sqrt{a^2 + b^2}$$

WHERE $P_{\bar{2}}$ = AXIAL LOAD IN BAR $\bar{2}$.

$P_{\bar{1}}$ = AXIAL LOAD IN BAR $\bar{1}$.

SIGN CONVENTION; + q CORRESPONDS TO SHEAR FLOW ON SIDE AB FROM A TO B.

+ P CORRESPONDS TO TENSION IN BAR.

FIGURE 4-7 SHEAR WEB REPLACEMENT WITH DIAGONALS

$$\begin{matrix}
 & u_{1p} & u_{2p} & u_{3p} & u_{1q} & u_{2q} & u_{3q} & u_{4p} & u_{5p} & u_{6p} & u_{4q} & u_{5q} & u_{6q} \\
 K_{6p} = & \begin{bmatrix}
 F_{1p} & & & & & & & & & & & & \\
 F_{2p} & C_{6p} & & & C_{6p} & & & & & 0 & & & 2C_{6p} \\
 F_{3p} & & & & & & & & & & & & \\
 F_{1q} & & & & & & & & & & & & \\
 F_{2q} & C_{6p} & & & C_{6p} & & & & & 0 & & & 2C_{6p} \\
 F_{3q} & & & & & & & & & & & & \\
 M_{4p} & & & & & & & & & & & & \\
 M_{5p} & & & & & & & & & & & & \\
 M_{6p} & 0 & & & 0 & & & & & 0 & & & 0 \\
 M_{4q} & & & & & & & & & & & & \\
 M_{5q} & & & & & & & & & & & & \\
 M_{6q} & 2C_{6p} & & & 2C_{6p} & & & & & 0 & & & 3C_{6p}
 \end{bmatrix}
 \end{matrix}$$

$$C_{6p} = \frac{1}{4 + \alpha_N}$$

$$\alpha_N = \frac{12 E I_N / l^3}{GA / l} K_N$$

FIGURE 4-8 SUBSTITUTE FACTORS IN K_S MATRIX TO ALLOW ELIMINATION OF ROTATIONAL RESTRAINT IN Z DIRECTION AT POINT P

$$\begin{matrix}
 & u_{1p} & u_{2p} & u_{3p} & u_{1q} & u_{2q} & u_{3q} & u_{4p} & u_{5p} & u_{6p} & u_{4q} & u_{5q} & u_{6q} \\
 K_{6q} = & \begin{bmatrix}
 F_{1p} & & & & & & & & & & & & \\
 F_{2p} & C_{6q} & & & C_{6q} & & & & & 2C_{6q} & & & 0 \\
 F_{3p} & & & & & & & & & & & & \\
 F_{1q} & & & & & & & & & & & & \\
 F_{2q} & C_{6q} & & & C_{6q} & & & & & 2C_{6q} & & & 0 \\
 F_{3q} & & & & & & & & & & & & \\
 M_{4p} & & & & & & & & & & & & \\
 M_{5p} & & & & & & & & & & & & \\
 M_{6p} & 2C_{6q} & & & 2C_{6q} & & & & & 3C_{6q} & & & 0 \\
 M_{4q} & & & & & & & & & & & & \\
 M_{5q} & & & & & & & & & & & & \\
 M_{6q} & 0 & & & 0 & & & & & 0 & & & 0
 \end{bmatrix}
 \end{matrix}$$

$$C_{6q} = \frac{1}{4 + a_N}$$

$$a_N = \frac{12 E I_N / l^3}{GA / l} K_N$$

FIGURE 4-9 SUBSTITUTE FACTORS IN K_S MATRIX TO ALLOW ELIMINATION OF ROTATIONAL RESTRAINT IN Z DIRECTION AT POINT Q

	u_{1p}	u_{2p}	u_{3p}	u_{1q}	u_{2q}	u_{3q}	u_{4p}	u_{5p}	u_{6p}	u_{4q}	u_{5q}	u_{6q}
--	----------	----------	----------	----------	----------	----------	----------	----------	----------	----------	----------	----------

$K_{5p} =$	<table style="width: 100%; border-collapse: collapse;"> <tr> <td style="padding: 5px;">F_{1p}</td> <td></td> </tr> <tr> <td style="padding: 5px;">F_{2p}</td> <td></td> </tr> <tr> <td style="padding: 5px;">F_{3p}</td> <td></td> <td></td> <td style="text-align: center;">C_{5p}</td> <td></td> <td></td> <td style="text-align: center;">C_{5p}</td> <td></td> <td style="text-align: center;">0</td> <td></td> <td></td> <td style="text-align: center;">$2C_{5p}$</td> <td></td> </tr> <tr> <td style="padding: 5px;">F_{1q}</td> <td></td> </tr> <tr> <td style="padding: 5px;">F_{2q}</td> <td></td> </tr> <tr> <td style="padding: 5px;">F_{3q}</td> <td></td> <td></td> <td style="text-align: center;">C_{5p}</td> <td></td> <td></td> <td style="text-align: center;">C_{5p}</td> <td></td> <td style="text-align: center;">0</td> <td></td> <td></td> <td style="text-align: center;">$2C_{5p}$</td> <td></td> </tr> <tr> <td style="padding: 5px;">M_{4p}</td> <td></td> </tr> <tr> <td style="padding: 5px;">M_{5p}</td> <td></td> <td></td> <td style="text-align: center;">0</td> <td></td> <td></td> <td style="text-align: center;">0</td> <td></td> <td style="text-align: center;">0</td> <td></td> <td></td> <td style="text-align: center;">0</td> <td></td> </tr> <tr> <td style="padding: 5px;">M_{6p}</td> <td></td> </tr> <tr> <td style="padding: 5px;">M_{4q}</td> <td></td> </tr> <tr> <td style="padding: 5px;">M_{5q}</td> <td></td> <td></td> <td style="text-align: center;">$2C_{5p}$</td> <td></td> <td></td> <td style="text-align: center;">$2C_{5p}$</td> <td></td> <td style="text-align: center;">0</td> <td></td> <td></td> <td style="text-align: center;">$3C_{5p}$</td> <td></td> </tr> <tr> <td style="padding: 5px;">M_{6q}</td> <td></td> </tr> </table>	F_{1p}													F_{2p}													F_{3p}			C_{5p}			C_{5p}		0			$2C_{5p}$		F_{1q}													F_{2q}													F_{3q}			C_{5p}			C_{5p}		0			$2C_{5p}$		M_{4p}													M_{5p}			0			0		0			0		M_{6p}													M_{4q}													M_{5q}			$2C_{5p}$			$2C_{5p}$		0			$3C_{5p}$		M_{6q}												
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M_{6q}																																																																																																																																																													

$$C_{5p} = \frac{1}{4 + a_T}$$

$$a_T = \frac{12 E I_T l^3}{GA l} K_T$$

FIGURE 4-10 SUBSTITUTE FACTORS IN K_S MATRIX ALLOWING ELIMINATION OF ROTATIONAL RESTRAINT IN Y DIRECTION AT POINT P

$$\begin{matrix}
 & u_{1p} & u_{2p} & u_{3p} & u_{1q} & u_{2q} & u_{3q} & u_{4p} & u_{5p} & u_{6p} & u_{4q} & u_{5q} & u_{6q} \\
 K_{5q} = & \left[\begin{array}{cccccccccccc}
 F_{1p} & & & & & & & & & & & & \\
 F_{2p} & & & & & & & & & & & & \\
 F_{3p} & & & C_{5q} & & & C_{5q} & & 2C_{5q} & & & 0 & \\
 F_{1q} & & & & & & & & & & & & \\
 F_{2q} & & & & & & & & & & & & \\
 F_{3q} & & & C_{5q} & & & C_{5q} & & 2C_{5q} & & & 0 & \\
 M_{4p} & & & & & & & & & & & & \\
 M_{5p} & & & 2C_{5q} & & & 2C_{5q} & & 3C_{5q} & & & 0 & \\
 M_{6p} & & & & & & & & & & & & \\
 M_{4q} & & & & & & & & & & & & \\
 M_{5q} & & & 0 & & & 0 & & 0 & & & 0 & \\
 M_{6q} & & & & & & & & & & & &
 \end{array} \right]
 \end{matrix}$$

$$C_{5q} = \frac{1}{4 + a_T}$$

$$a_T = \frac{12E I_T / l^3}{GA/l} K_T$$

FIGURE 4-11 SUBSTITUTE FACTORS IN K_S MATRIX ALLOWING ELIMINATION OF ROTATIONAL RESTRAINT IN Y DIRECTION AT POINT Q

where $[\lambda]$ is a matrix of direction cosines (cosines of angles between Local and Global Coordinate Systems) obtained by resolving global displacements in the direction of local coordinates. For the general bar, $[\lambda]$ is given in Equation (4-3).

$$\lambda = \begin{bmatrix} \bar{\lambda} & 0 & 0 & 0 \\ 0 & \bar{\lambda} & 0 & 0 \\ 0 & 0 & \bar{\lambda} & 0 \\ 0 & 0 & 0 & \bar{\lambda} \end{bmatrix} \quad (4-3)$$

In Equation (4-3), each $\bar{\lambda}$ is a 3 by 3 matrix of the direction cosines of the local coordinate axes relative to the global system, as shown in Equation (4-4).

$$\bar{\lambda} = \begin{bmatrix} l_1 & m_1 & n_1 \\ l_2 & m_2 & n_2 \\ l_3 & m_3 & n_3 \end{bmatrix} \quad (4-4)$$

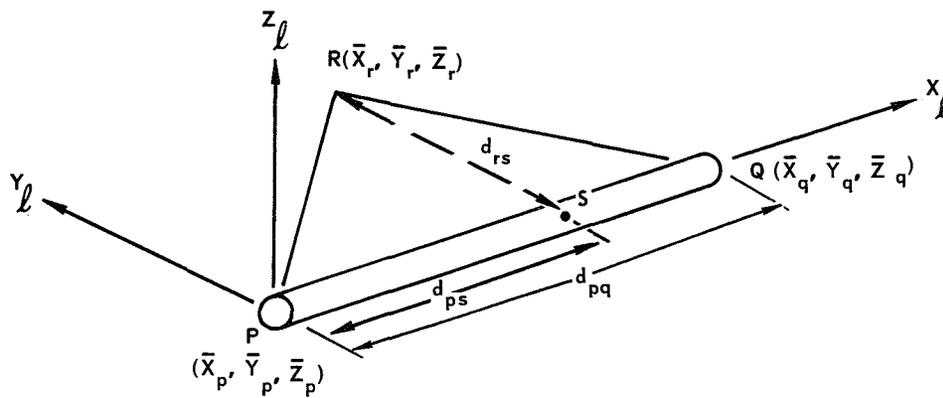
The row format of λ corresponds to the sequence of displacements in the Local Coordinate System specified for the stiffness matrix in Figure 4-4. The column format of λ corresponds to a similar sequence in the Global Coordinate System (i.e., $\bar{u}_1, \bar{u}_2, \dots, \bar{u}_6$). Values of the direction cosines relating the Local Coordinate System to the Global Coordinate System are determined as indicated in Figure 4-12.

Each element stiffness matrix is transformed from the Local to the Global Coordinate System by Equation (4-5)

$$[\bar{K}] = [\lambda]^T [K] [\lambda] \quad (4-5)$$

where $[\bar{K}]$ is the element stiffness matrix transformed to Global Coordinate System and $[\lambda]^T$ is the transpose of transformation matrix $[\lambda]$.

The total stiffness matrix (K_A) for the assembled structure (in the Global Coordinate System) as shown in Figure 4-13, is generated by systematically adding the transformed element stiffness matrices. Nodal points on



$$d_{pq} = \sqrt{(\bar{X}_q - \bar{X}_p)^2 + (\bar{Y}_q - \bar{Y}_p)^2 + (\bar{Z}_q - \bar{Z}_p)^2}$$

$$d_{rs} = \sqrt{(\bar{X}_r - \bar{X}_p)^2 + (\bar{Y}_r - \bar{Y}_p)^2 + (\bar{Z}_r - \bar{Z}_p)^2} - d_{ps}^2$$

$$d_{ps} = l_1 (\bar{X}_r - \bar{X}_p) + m_1 (\bar{Y}_r - \bar{Y}_p) + n_1 (\bar{Z}_r - \bar{Z}_p)$$

DIRECTION COSINE	COSINE OF ANGLE BETWEEN		EQUATION FOR DIRECTION COSINE
	LOCAL AXIS	GLOBAL AXIS	
l_1	x_l	\bar{X}	$(\bar{X}_q - \bar{X}_p)/d_{pq}$
m_1	x_l	\bar{Y}	$(\bar{Y}_q - \bar{Y}_p)/d_{pq}$
n_1	x_l	\bar{Z}	$(\bar{Z}_q - \bar{Z}_p)/d_{pq}$
l_2	y_l	\bar{X}	$[(\bar{X}_r - \bar{X}_p) - l_1 d_{ps}]/d_{rs}$
m_2	y_l	\bar{Y}	$[(\bar{Y}_r - \bar{Y}_p) - m_1 d_{ps}]/d_{rs}$
n_2	y_l	\bar{Z}	$[(\bar{Z}_r - \bar{Z}_p) - n_1 d_{ps}]/d_{rs}$
l_3	z_l	\bar{X}	$m_1 n_2 - m_2 n_1$
m_3	z_l	\bar{Y}	$-l_1 n_2 + l_2 n_1$
n_3	z_l	\bar{Z}	$l_1 m_2 - l_2 m_1$

NOTE: POINT R IS AN ARBITRARILY SELECTED POINT LYING IN THE $x_l - y_l$ PLANE. IT IS USED TO IDENTIFY THE ORIENTATION OF MEMBER BENDING AXES z_l AND y_l RELATIVE TO THE GLOBAL COORDINATE SYSTEM. (I_N IS THE MOMENT OF INERTIA ABOUT THE z_l AXIS AND I_T IS THE MOMENT OF INERTIA ABOUT THE y_l AXIS.)

FIGURE 4-12 DIRECTION COSINES OF LOCAL COORDINATE SYSTEM RELATIVE TO GLOBAL COORDINATE SYSTEM

		1			• •			n _i																							
		t			r			• • • •			t			r																	
		\bar{x}	\bar{y}	\bar{z}	\bar{x}	\bar{y}	\bar{z}	••••••••	\bar{x}	\bar{y}	\bar{z}	\bar{x}	\bar{y}	\bar{z}	\bar{x}	\bar{y}	\bar{z}														
1	t	\bar{x}	<p>SYMMETRIC</p> <p>(THE ELEMENTS OF THE MATRICES OF INDIVIDUAL ELEMENT STIFFNESSES IN THE GLOBAL COORDINATE SYSTEM ARE SUCCESSIVELY ADDED INTO THE APPROPRIATE LOCATIONS IN THE FRAMEWORK OF THIS MATRIX. THE KNOWN SYMMETRY OF THE MATRIX IS UTILIZED IN ITS GENERATION.)</p>																												
		\bar{y}																													
		\bar{z}																													
	r	\bar{x}																													
		\bar{y}																													
		\bar{z}																													
•	•	•																•	•	•	•	•	•	•	•	•	•	•	•	•	•
	•	•																•	•	•	•	•	•	•	•	•	•	•	•	•	•
	•	•																•	•	•	•	•	•	•	•	•	•	•	•	•	•
n _i	t	\bar{x}																													
		\bar{y}																													
		\bar{z}																													
	r	\bar{x}																													
		\bar{y}																													
		\bar{z}																													

SYMBOLS t AND r INDICATE TRANSLATIONAL AND ROTATIONAL DISPLACEMENTS RESPECTIVELY.

FIGURE 4-13 ASSEMBLED STIFFNESS MATRIX FORMAT

the idealized structure are numbered consecutively from 1 to n_j . The stiffness matrix K_A is assembled with a row and column format corresponding to the three translational followed by the three rotational degrees of freedom in the global system at each node in sequence. In this general case, the size of the stiffness matrix is $6n_j \times 6n_j$.

4.1.1.3 Elastic Analysis - The assembled stiffness matrix $[K_A]$ is related to the column matrices of global forces and displacements at each node by Equation (4-6) which represents a combination of Equation (4-1) through (4-5). The word "forces" in this discussion implies both forces and moments, and the word "displacements" implies both deflections and rotations.

$$\{F\} = [K_A]\{\delta_G\} \quad (4-6)$$

The stiffness matrix is singular. That is, its determinant vanishes and its inverse does not exist. Boundary conditions (supports) must be defined to prevent rigid body motion. Once $[K_A]$ has been determined, a solution can be obtained for any set of support conditions.

As described in Reference (3), the unknown nodal displacements and support reactions are normally obtained by partitioning Equation (4-6) according to the location and orientation of supports as indicated in Equation (4-7).

$$\begin{Bmatrix} F_{(m-n)} \\ F_n \end{Bmatrix} = \begin{bmatrix} A_{(m-n) \times (m-n)} & B_{(m-n) \times n} \\ B'_n \times (m-n) & D_{n \times n} \end{bmatrix} \begin{Bmatrix} \delta_{(m-n)} \\ \delta_n \end{Bmatrix} \quad (4-7)$$

Subscripts n and m are

n = number of support boundary conditions (i.e., $\delta_n = 0$),

m = order of stiffness matrix.

This partitioning results in two sets of equations: Equation (4-8) relates unknown nodal displacements to known applied forces, and Equation (4-9) relates unknown support reactions to unknown nodal displacements.

$$\{F_{(m-n)}\} = [A]\{\delta_{(m-n)}\} \quad (4-8)$$

$$\{F_n\} = [B^T]\{\delta_{(m-n)}\} \quad (4-9)$$

The inverse of matrix [A] in Equation (4-8) is the flexibility matrix of the structure. Equation (4-8) may be rewritten to give unknown nodal displacements in terms of the flexibility matrix and known applied forces as shown in Equation (4-10).

$$\{\delta_{(m-n)}\} = [A]^{-1}\{F_{(m-n)}\} \quad (4-10)$$

The unknown reactions may be obtained by combining Equations (4-9) and (4-10) as shown in Equation (4-11).

$$\{F_n\} = [B^T][A]^{-1}\{F_{m-n}\} \quad (4-11)$$

The preceding discussion outlines the method normally used to obtain nodal displacements and reactions. Due to computer core limitations and the desire to minimize program running time, an iterative method of determining nodal deflections and reactions, once the stiffness matrix was established, was programmed. An iterative method eliminates the need for matrix inversion.

The requirement for plastic supports calls for multiple solutions of sets of equations very nearly the same. These sets of simultaneous equations are represented by matrix Equation (4-8), relating unknown nodal deflections to known applied forces. This type of problem is best handled by an iterative solution, since the required nodal deflections are approximately known after the initial elastic solution is determined, thus minimizing computing time. Four iterative techniques were programmed; Gauss-Siedel, Gauss-Siedel with Aitkens Delta Squared improvements, the Overrelaxation method, and the Conjugate Gradient technique. Reference (4) describes the first three methods and Reference (5) the latter. Any one of these techniques may be best suited for a specific structural problem. However, experience dictates that for highly redundant space frames, such as the legged lander, the Conjugate Gradient technique appears to be the best suited of the above solution methods.

After nodal displacements have been determined using the iteration procedure, the unknown support reactions are obtained by substitution of these nodal displacements into matrix Equation (4-9). Forces on elements in the Local Coordinate System are found by transforming the nodal displacements into the Local Coordinate System using Equation (4-2) and applying the appropriate force-displacement relationships using Equation (4-1).

The program also is capable of solving problems wherein the nodal displacements are known, as in problems where some supports settle. Combinations of known applied forces and known nodal displacements may be input, and all forces and displacements will be determined. Boundary conditions imposed on the problem must be sufficient to prevent rigid body motion. Partitioning of Equation (4-7) for this case results in Equation (4-12), relating unknown nodal displacements to known applied forces and known nodal displacements, and in Equation (4-13), which relates unknown forces and reactions to the now known (determined in Equation (4-12)) nodal displacements. Subscript n for this case implies either zero or nonzero known boundary conditions.

$$\{F_{(m-n)}\} = [A]\{\delta_{(m-n)}\} + [B]\{\delta_n\} \quad (4-12)$$

$$\{F_n\} = [B']\{\delta_{(m-n)}\} + [D]\{\delta_n\} \quad (4-13)$$

This solution is simply a more general case of the initial partitioning (Equation (4-8) and (4-9)), where the only known nodal displacements were zero and terms involving δ_n were therefore not included.

4.1.1.4 Plastic Support - When analyzing a space frame it may be desirable to idealize support members with restricted load-carrying capability. An upper (positive) and lower (negative) limit can be set on the magnitude of any force or moment component in such a support member. Initial solution of the space frame redundancy assumes elastic deformation, but the magnitude of the load components at the support are compared with input limits to ascertain if these have been exceeded. Those load components (reactions) which exceed the input limits are then assumed to be at the input cutoff values. The new set of boundary conditions (new column matrix of deflections and forces) are

then employed with the stiffness matrix to obtain a new solution. The load components at the supports and the limits are again compared. If the limits are exceeded, the process is repeated until no limits are exceeded and equilibrium is achieved.

A plastic attenuator support can be modeled with this feature by setting the appropriate input limit to the force causing plastic deformation. Cables with one end attached to a support and the other to a structural joint can be idealized by making one of the limits on each reaction force component zero, allowing the cable to carry only tension and no compression. Determination of the proper limit (positive or negative) to use on a particular force component depends on whether the desired limit is in the positive or negative global coordinate direction. If a desired tension force in a cable results in a force component in the positive global direction at the support, the lower (negative) limit is set equal to zero. If the desired tension force component is in the negative global direction, the upper (positive) limit is set equal to zero.

4.1.1.5 Modal Analysis - The frequencies and mode shapes for the free-free center body structure are determined once the unrestrained stiffness matrix of the center body (Section 4.1.1.2) has been obtained. This modal analysis is performed in the optional Modal Analysis Routine in the Center Body Option of the Structural Analysis Program.

Free vibrations of the center body structure are defined by Equation (4-14)

$$[M]\{\ddot{q}(s, t)\} + [K]\{q(s, t)\} = 0 \quad (4-14)$$

where

[M] = mass matrix of the center body.

[K] = stiffness matrix of the center body.

$\{\ddot{q}\}, \{q\}$ = accelerations and displacements describing the motion of the control points throughout the center body.

s = space coordinates in the center body.

t = time.

The above representation of the stiffness and inertia characteristics of the center body are input data for the Modal Analysis Routine. An eigenvalue routine is used to obtain the vibratory free-free mode shapes and corresponding frequencies.

The large order center body stiffness matrix results in an eigenvalue problem which is too large to solve practically. The computer run time required to obtain the frequencies and mode shapes would be excessive. In addition, the eigenvalue problem would exceed the allotted computer core storage requirements. For these reasons, the size of the center body's structural stiffness matrix is reduced before solving the eigenvalue problem. This reduction technique is discussed in Reference (6).

In the reduction procedure, a number of degrees of freedom, corresponding to various displacements and rotations at the center body joints, are removed. To remove these, it is assumed that the inertia forces and/or moments associated with these degrees of freedom are negligible. In this procedure, all of the strain energy associated with the removed degrees of freedom is retained. This reduction procedure is outlined as follows.

$$\begin{Bmatrix} P \\ 0 \end{Bmatrix} = \begin{bmatrix} K_{11} & K_{12} \\ K_{21} & K_{22} \end{bmatrix} \begin{Bmatrix} q_1 \\ q_2 \end{Bmatrix} \quad (4-15)$$

where

K_{11} , K_{12} , K_{21} , K_{22} = segments of total center body stiffness matrix.

q_1 = degrees of freedom on which forces and moments (P's) exist.

q_2 = degrees of freedom on which negligible forces and moments exist.

In the above, the total stiffness matrix has been reordered such that the elements associated with the degrees of freedom to be retained appear first.

Equation (4-15) is equivalent to the two following expressions

$$\{P\} = [K_{11}]\{q_1\} + [K_{12}]\{q_2\} \quad (4-16)$$

$$\{0\} = [K_{21}]\{q_1\} + [K_{22}]\{q_2\} \quad (4-17)$$

Solving the second of these for $\{q_2\}$ gives

$$\{q_2\} = -[K_{22}]^{-1}[K_{21}]\{q_1\} \quad (4-18)$$

Substituting this into Equation (4-16) results in the reduced form

$$\{P\} = \left[[K_{11}] - [K_{12}][K_{22}]^{-1}[K_{21}] \right] \{q_1\} \quad (4-19)$$

from which the reduced stiffness matrix, $[K^*]$, is defined as

$$[K^*] = [K_{11}] - [K_{12}][K_{22}]^{-1}[K_{21}] \quad (4-20)$$

The eigenvalue problem associated with the reduced system is

$$[M^*]\{\ddot{q}_1\} + [K^*]\{q_1\} = 0 \quad (4-21)$$

where $[M^*]$ is a diagonal mass matrix whose elements represent the distribution of the center body mass at the degrees of freedom to be retained. Frequencies and mode shapes of the total elastic center body structure are obtained using Equations (4-18) and (4-21). Reference (7) summarizes the Householder-Ortega-Wilkinson Method used to determine mode shapes and natural frequencies. An example of the use of this routine is given in Section 4.3.1.3.

4.1.2 Landing Gear Option - The Landing Gear Option of the Structural Analysis Program can be used to investigate the energy absorption characteristics and internal loads in inverted tripod or cantilever landing gears. The finite element stiffness method employed in this option of the program is based on large displacement (nonlinear) finite element theory.

In linear finite element theory, changes in the stiffness matrix due to displacements are assumed to be negligible. Landing gear struts experience large rotational and extensional displacements during stroking of the gear. These displacements require that the finite element idealization of the struts include "rotational" and "extensional" nonlinearities. To illustrate the nonlinear nature of the problem, consider a planar gear subjected to the displacements shown in Figure 4-14. Stiffness of the gear changes appreciably during stroking from the original to the displaced position. This change is due to the following displacement nonlinearities:

- (1) rotational nonlinearities - large rotations cause significant changes in strut orientation and the resulting stiffness;
- (2) extensional nonlinearities - compressive (or tensile) strut crushing changes the strut length and slope of the axial load-stroke curve (referred to as axial stiffness) both of which alter the stiffness.

For these reasons linear theory cannot be used in the analysis of landing gears since changes in the stiffness matrix due to displacements are not negligible.

4.1.2.1 Modified Incremental Stiffness Method - To account for the nonlinearities associated with landing gear analysis, the incremental stiffness finite element method is employed. A modification of this method is made to insure that load unbalances do not result in the gear due to the linear approximations made in each step of the solution technique.

In the incremental stiffness method, nonlinear behavior is approximated through a sequence of linear solutions. The loading is divided into a number of incremental steps. For each step, an increment of the external load is applied and incremental displacements determined. These displacements, when added to the structural positions at the conclusion of the previous step, define updated geometry. The incremental stiffness matrix, used to determine these incremental displacements, is updated at the conclusion of each step to reflect changes due to displacement nonlinearities. Because small increments

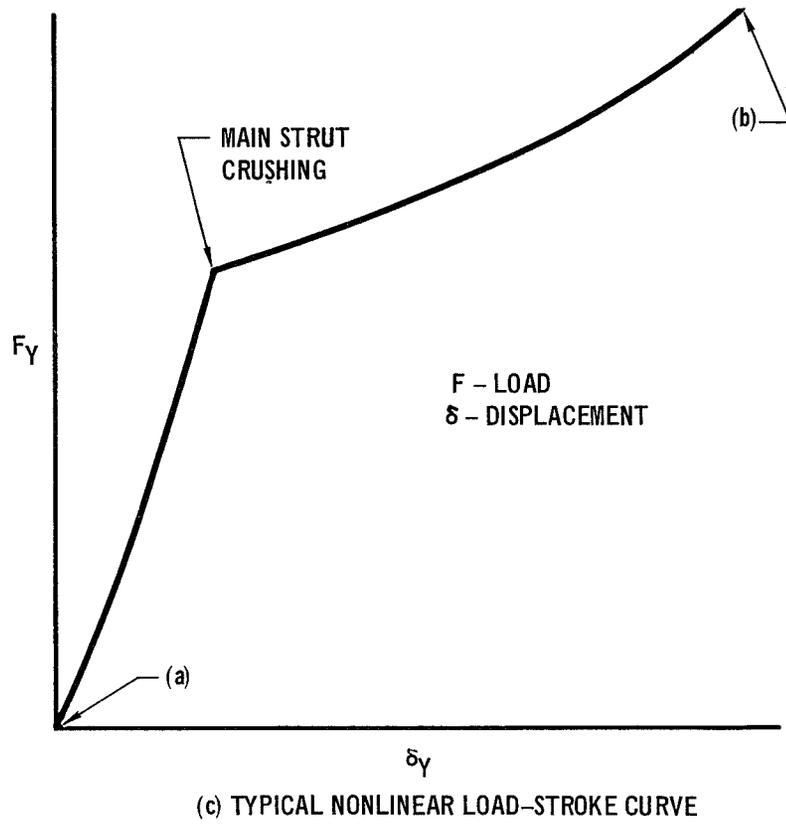
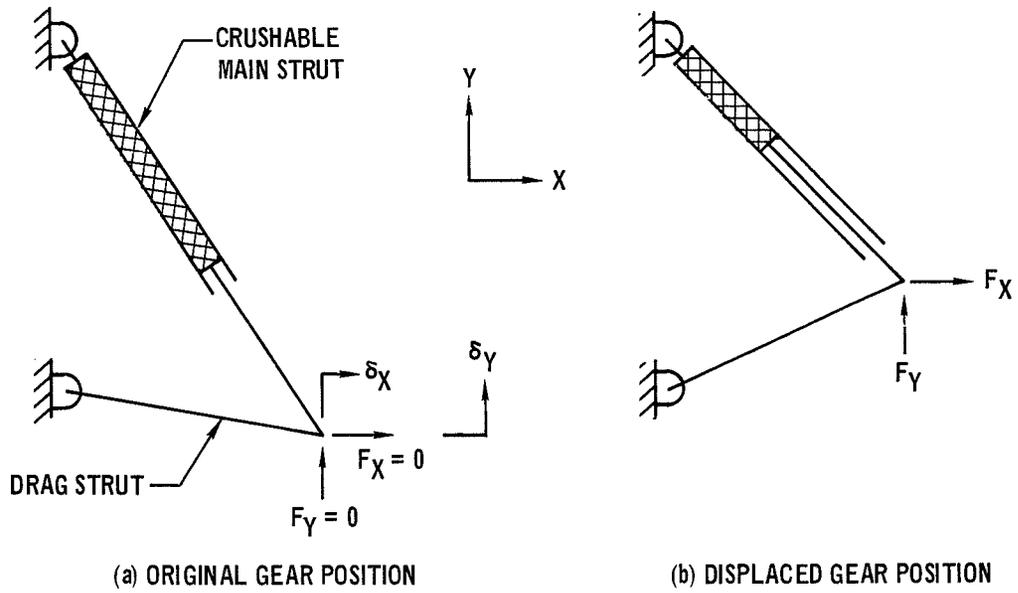


FIGURE 4-14 LARGE DISPLACEMENTS OF A TYPICAL TWO-DIMENSIONAL GEAR

are used, a linear small displacement problem is solved at each step. The method is illustrated in Figure 4-15 for a typical nonlinear load-stroke curve. For each step n , the incremental displacement $(\delta_n - \delta_{n-1})$ corresponding to the applied incremental load $(F_n - F_{n-1})$ is determined from the equation

$$\delta_n = \delta_{n-1} + (F_n - F_{n-1})/S_n, \quad n = 1, \dots, m \quad (4-22)$$

where m is the total number of incremental steps. In this equation, S_n is the incremental stiffness of the structure corresponding to the displaced state, δ_{n-1} . For example, at the conclusion of step 1 (point A_1 in Figure 4-15), the incremental stiffness S_2 (at point P_1) is determined for the displaced state, δ_1 . This stiffness and the next incremental load $(F_2 - F_1)$ are then used in Equation (4-22) to determine δ_2 (defining point A_2),

$$\delta_2 = \delta_1 + (F_2 - F_1)/S_2 \quad (4-23)$$

This process is repeated until all load increments have been applied. For the gear of Figure 4-14 the above method would be applied simultaneously to the nonlinear load-stroke curves in the X and Y directions.

Due to the linear approximation at each step of the incremental stiffness method, error in the solution accumulates as a function of step size. This can be seen in Figure 4-15 where at the conclusion of step 4 the difference between points A_4 and P_4 is significant. This error, the difference between the applied load (F_4) and the true load corresponding to the displaced state (the force at point P_4), is defined as the load unbalance. To eliminate this unbalance, the incremental stiffness method is combined with an iteration procedure to assure convergence to the correct solution. For each incremental step, iteration is employed to insure that the internal loads, corresponding to the displaced state, are in equilibrium with the applied loads.

The iteration approach as applied to the n th incremental step of Figure 4-15 is illustrated in Figure 4-16. The incremental displacement $\delta_n - \delta_{n-1}$, corresponding to the incremental applied load $F_n - F_{n-1}$, is sought. Knowing the incremental stiffness S_1 at the beginning of the n th incremental step, the first estimate of the incremental displacement, $\Delta\delta_1$, is found from

$$\Delta\delta_1 = (F_n - F_{n-1})/S_1 \quad (4-24)$$

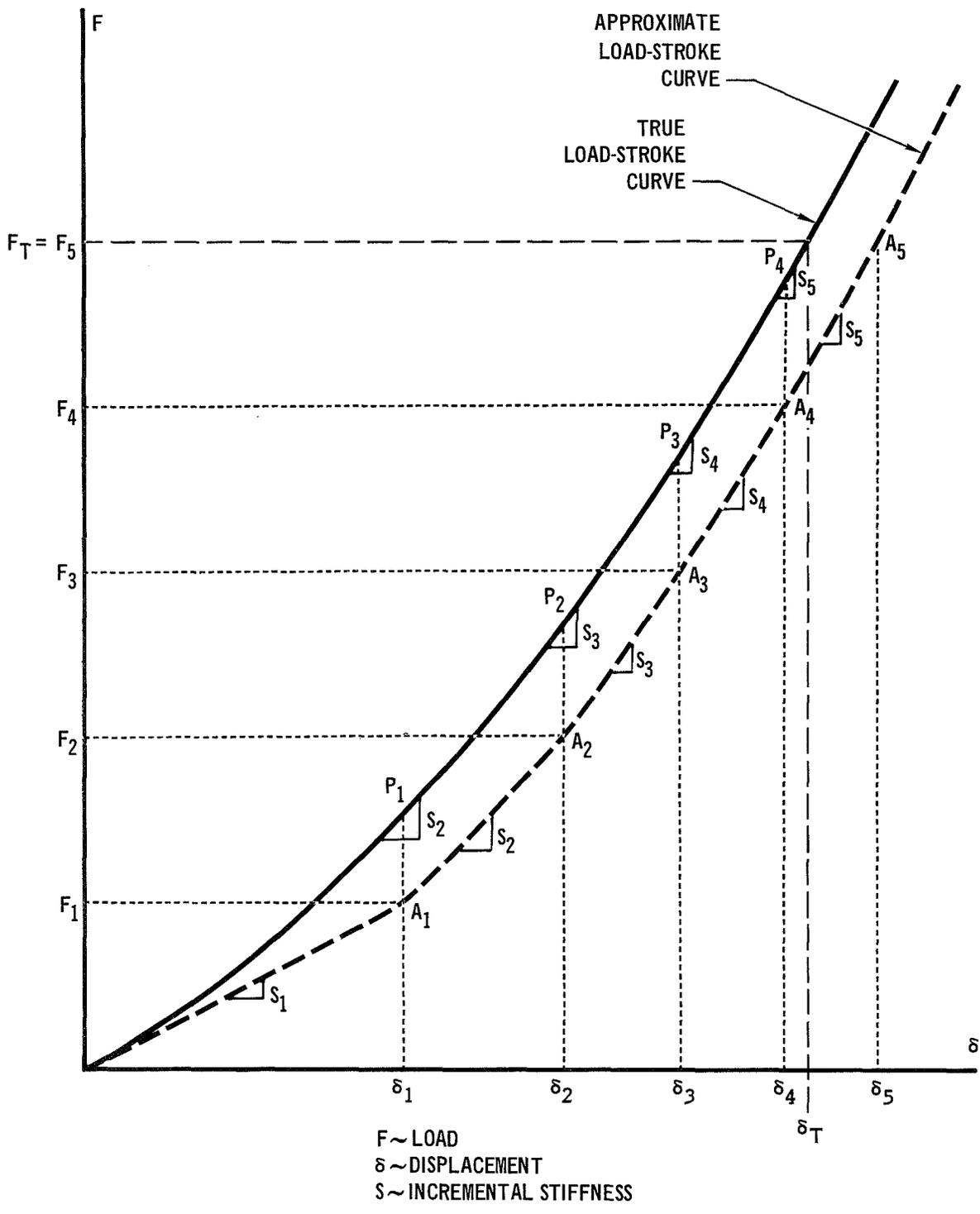


FIGURE 4-15 INCREMENTAL STIFFNESS METHOD APPLIED TO A TYPICAL NONLINEAR LOAD-STROKE CURVE

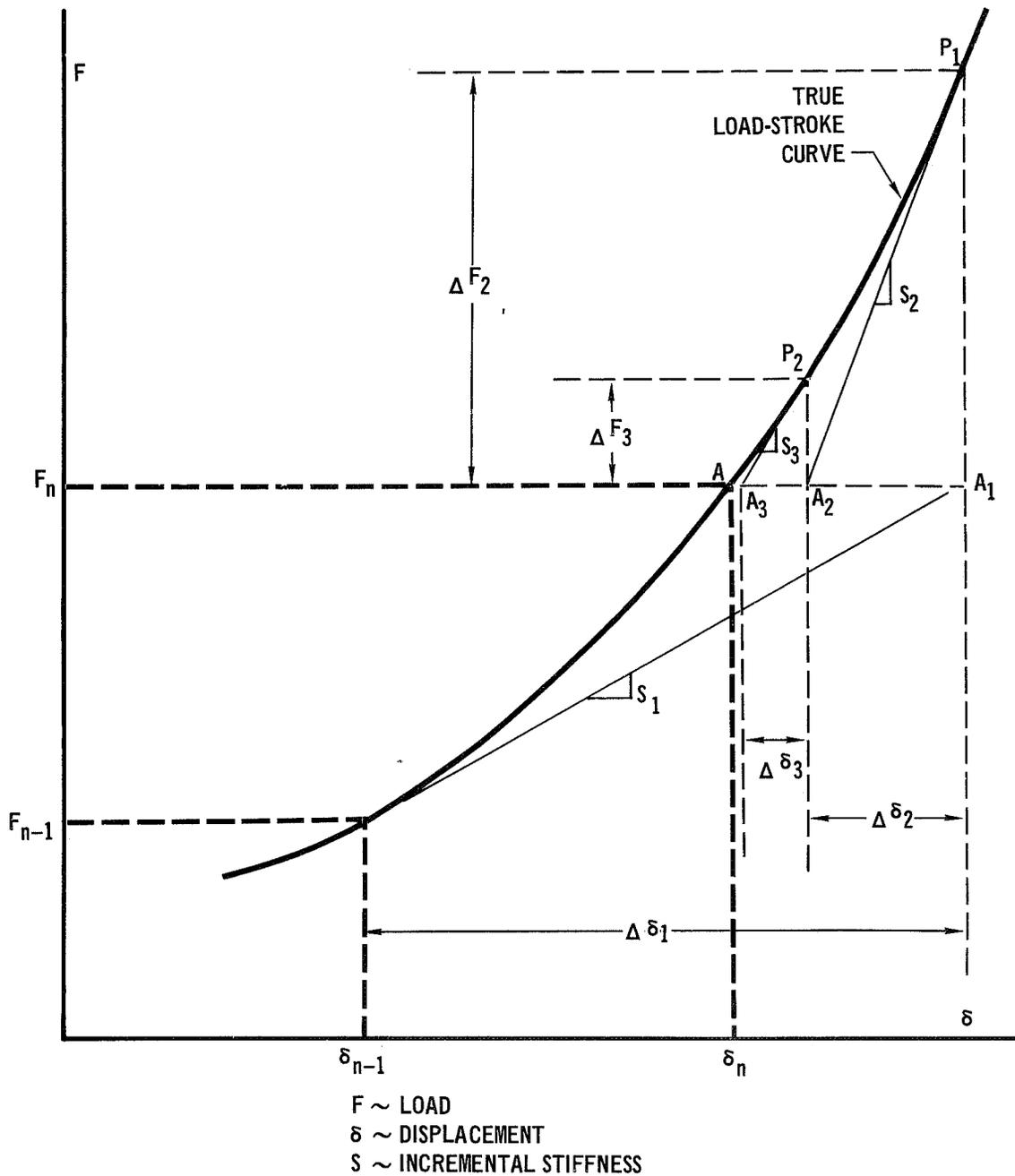


FIGURE 4-16 TYPICAL INCREMENTAL STEP OF MODIFIED INCREMENTAL STIFFNESS METHOD

This yields point A_1 . The true incremental stiffness S_2 and the resultant strut reaction load $F_n + \Delta F_2$ corresponding to the displacement state $\delta_{n-1} + \Delta\delta_1$ are defined by point P_1 . The force unbalance ΔF_2 , the difference between the total applied load, F_n , and the reaction load $F_n + \Delta F_2$, is applied to the structure whose incremental stiffness is S_2 . Thus, the new estimate of the incremental displacement, $\Delta\delta_1 + \Delta\delta_2$, is determined where $\Delta\delta_2$ is defined by

$$\Delta\delta_2 = \Delta F_2 / S_2 \quad (4-25)$$

This process is repeated and a new estimate of the incremental displacement, $\sum_{i=1}^{\ell} \Delta\delta_i$, is determined using the relation

$$\Delta\delta_{\ell} = \Delta F_{\ell} / S_{\ell}, \quad \ell = 3, \dots \quad (4-26)$$

until ΔF_{ℓ} is arbitrarily small (convergence at point A). When this occurs, the applied load is balanced by the reaction load and equilibrium has been attained.

For the gear of Figure 4-14, the above method would be applied simultaneously to the nonlinear load-stroke curves in the X and Y directions. For each of these directions, the force unbalance would be the difference between the applied load in that direction and the sum of the internal drag strut and main strut loads in that direction.

The modified incremental stiffness method is well suited to the analysis of landing gears. When employing this method, errors will not be introduced when structural stiffness properties change abruptly as is common in landing gear members containing attenuation. The error at each step of the process is known since the system is in equilibrium within a predetermined tolerance. In addition, the method provides a displacement-load history for any desired number of increments of applied gear stroke. This is essential for determining energy absorption characteristics of landing gears.

The modified incremental stiffness method, as applied to three dimensional inverted tripod or cantilever gears, requires the solution of the matrix equation

$$\Delta\{F\} = [S] \Delta\{\delta\} \quad (4-27)$$

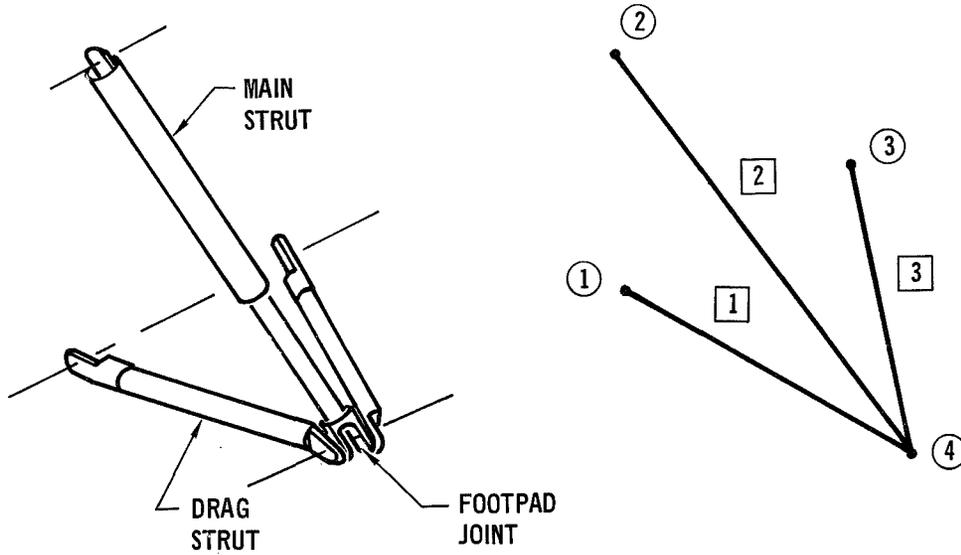
Equation (4-27) is solved several times for each step of the loading. In this equation, $\Delta\{F\}$ is a column matrix of incremental forces applied at the nodes of the gear; $\Delta\{\delta\}$ is a column matrix of incremental nodal displacements; and $[S]$ is the instantaneous incremental stiffness of the gear. For landing gear analysis, the independent variables in Equation (4-27) for each step of the process will be the applied displacements. The method of employing Equation (4-27) for each step of the process is as follows:

- (i) Equation (4-27) is solved once for the case where $\Delta\{\delta\}$ is the total applied displacement vector divided by the number of incremental steps. The incremental stiffness matrix $[S]$ of the gear corresponds to the displaced conditions existing at the conclusion of the previous step. This resulting equation is analogous to Equation (4-22). The nodal locations are then updated.
- (ii) Equation (4-27) is then solved repeatedly (iteration is employed) until the largest load unbalance component is less than a predetermined tolerance. At the conclusion of each iteration, nodal locations are updated. For the first of these iterations, $[S]$ corresponds to the displaced condition existing at the conclusion of (i). For all succeeding iterations, $[S]$ corresponds to the displaced condition existing at the conclusion of the previous iteration. $\Delta\{F\}$ is a vector representing the difference between the applied load components and the components of internal load corresponding to the displaced condition existing at the conclusion of the previous iteration. Hence, $\Delta\{F\}$ represents the load unbalances. Each of the above equations is analogous to Equation (4-26).

4.1.2.2 Structural Idealization - The Landing Gear Option employs a fixed idealization for inverted tripod and cantilever landing gears. This idealization (node point and element numbering) must be adhered to when employing this option of the program.

The inverted tripod landing gear is idealized in the program with four nodes and three pin-ended elements (the main strut and two drag struts) as shown in Figure 4-17(a). These elements are capable of carrying axial loads only and may contain honeycomb attenuation for both tension and compression. Each element may be made of a different material. Each node is assumed to be

(a) INVERTED TRIPOD GEAR



NOTE:

② JOINT NUMBER

4 ELEMENT NUMBER

(b) CANTILEVER GEAR

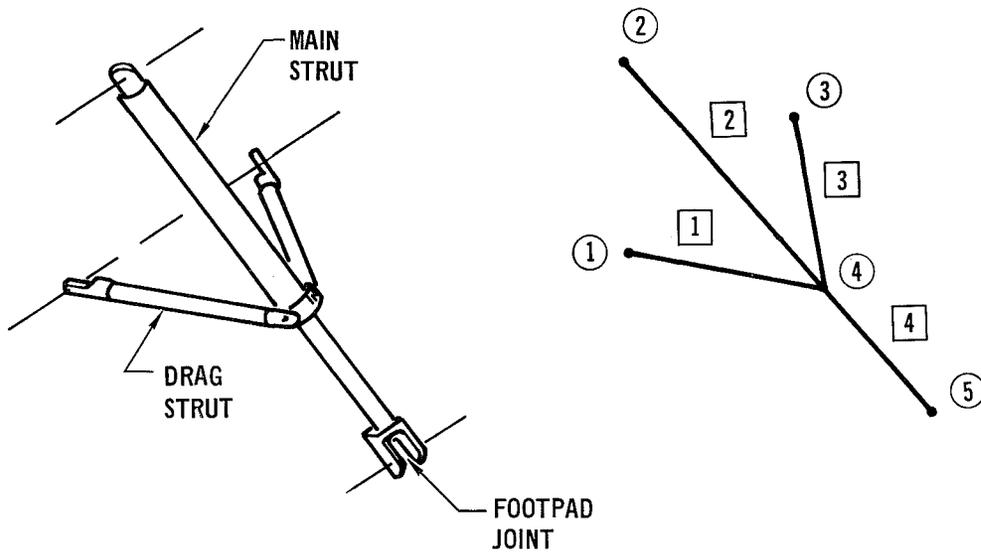


FIGURE 4-17 LANDING GEAR IDEALIZATIONS

a pin which can support three force components but no moments. Each element of a gear has associated with it a set of three nodes (p, q, and r) which defines the Local Coordinate System, as explained in Section 4.1.2.3, for that element. The fixed idealization of the inverted tripod gear employs the following nodal point numbering system for the elements:

Element				
No.	p	q	r	
1	1	4	2	
2	2	4	3	
3	3	4	2	

The cantilever landing gear is idealized with five nodes and four elements as shown in Figure 4-17(b). The main strut is idealized with two elements, both of which are capable of carrying bending as well as axial loads. These members (elements 2 and 4 in Figure 4-17(b)) are assumed to be rigidly connected at node 4. This provides moment continuity along the main strut. For elements 2 and 4 of the main strut, the moment of inertia about any axis normal to the element is assumed to be constant. However, the moment of inertia for element 2 may be different than that for element 4. For each of these elements, the modulus of elasticity for bending displacements may be different than the modulus for axial displacements. The junctions of the main strut with the center body (node 2) and the footpad (node 5) are assumed to be pinned. The drag struts (elements 1 and 3) are pin-ended with axial load-carrying capability only. Thus, a drag strut cannot carry bending moments at either end. Both drag struts (elements 1 and 3) and the lower element of the main strut (element 4) may contain honeycomb attenuation for both tension and compression. Each strut of the cantilever gear may be made of a different material. The fixed idealization of the cantilever gear employs the following nodal point numbering system for the elements:

Element				
No.	p	q	r	
1	1	4	2	
2	2	4	RP	
3	3	4	2	
4	5	4	RP	

Point r (RP) for elements 2 and 4 is a floating reference point whose coordinates are continually changing. Initially, this point is selected as node 1. As the main strut bends, the floating reference point is located as described in Section 4.1.2.5.

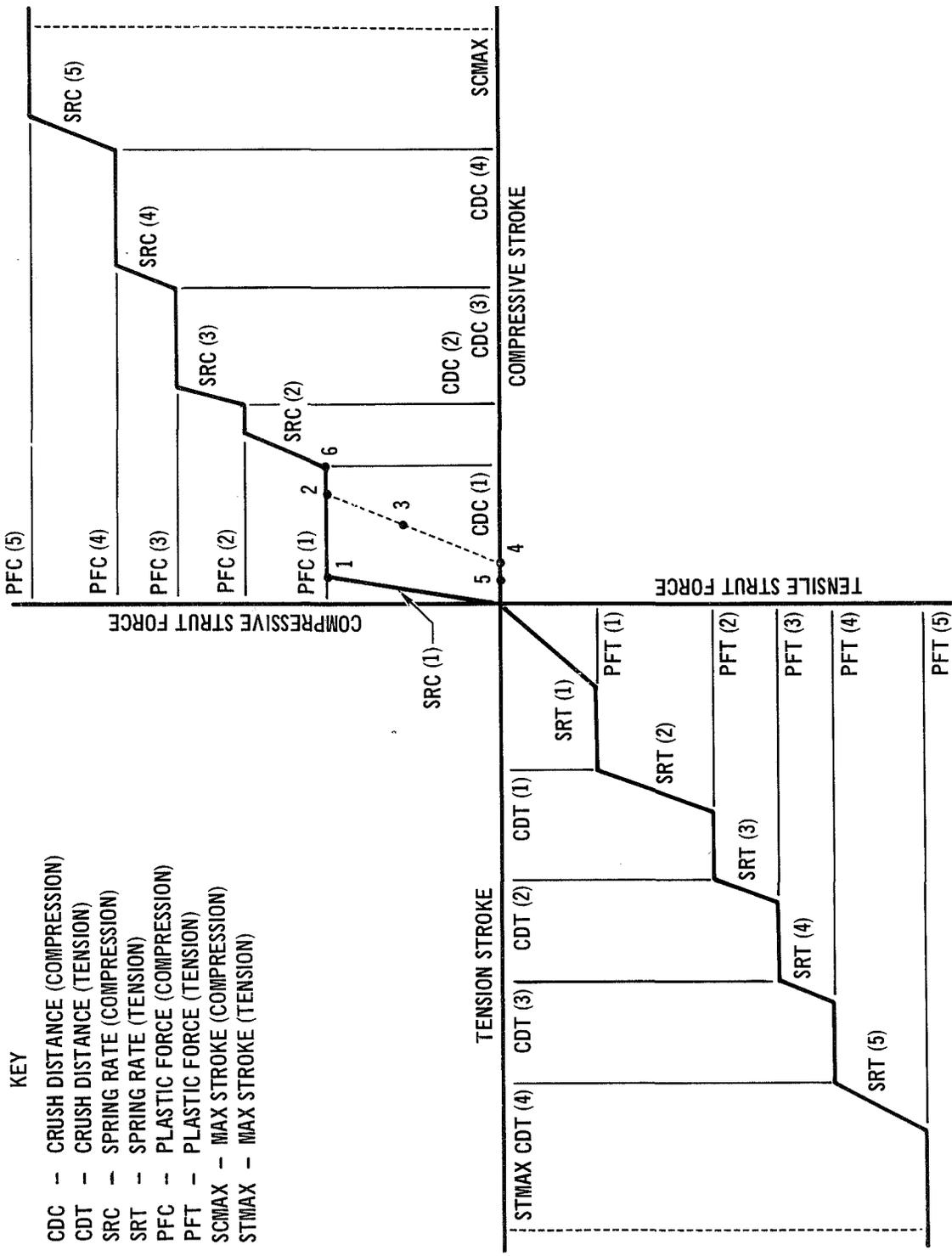
For either of the above gears, struts which can carry axial loads only (all elements of the inverted tripod gear and elements 1 and 3 of the cantilever gear) are defined as "axial struts." Struts which are capable of carrying bending as well as axial loads (elements 2 and 4 of the cantilever gear) are defined as "bending struts."

For a strut which contains honeycomb attenuation, the material defining the strut properties is assumed to have a load-stroke curve for axial displacements similar to that shown in Figure 4-18. This curve represents the stroking characteristics of stacked honeycomb cartridges housed within a landing gear strut. Each cartridge is assumed to crush at constant load as the strut is stroked. Cartridges possessing different crushing strengths may be stacked in series to form a desired load-stroke characteristic. Up to five cartridges can be used to attenuate compression loads and up to five to attenuate tension loads.

For the typical load-stroke curve shown in Figure 4-18, determination of the axial load corresponding to stroking causing compression in the strut is explained in the following discussion. The axial load corresponding to tensile stroking of the strut would be determined in a similar manner.

As the strut initially begins to stroke, the strut load increases linearly with stroke to point 1 where the first crush load is reached. The load then remains constant with stroke until either the stroke reverses direction or a second elastic portion is reached. If the direction of stroke reverses, point 2, one of the following load-stroke sequences is possible:

- (1) Elastic unloading to an intermediate point between 2 and 4, such as point 3, at which time the compressive stroke again increases. This results in the load increasing elastically to point 2 and then following the original load-stroke curve.
- (2) Elastic unloading through point 3 to point 4. A continued decrease in stroke to point 5 occurs at a zero strut load. With a reversal of stroke the strut will compress with zero load until point 4 is



- KEY
- CDC - CRUSH DISTANCE (COMPRESSION)
 - CDT - CRUSH DISTANCE (TENSION)
 - SRC - SPRING RATE (COMPRESSION)
 - SRT - SPRING RATE (TENSION)
 - PFC - PLASTIC FORCE (COMPRESSION)
 - PFT - PLASTIC FORCE (TENSION)
 - SCMAX - MAX STROKE (COMPRESSION)
 - STMAX - MAX STROKE (TENSION)

FIGURE 4-18 TYPICAL STRUT LOAD-STROKE CURVE

again reached. The load will then increase linearly to point 2 and continue to follow the original load-stroke curve.

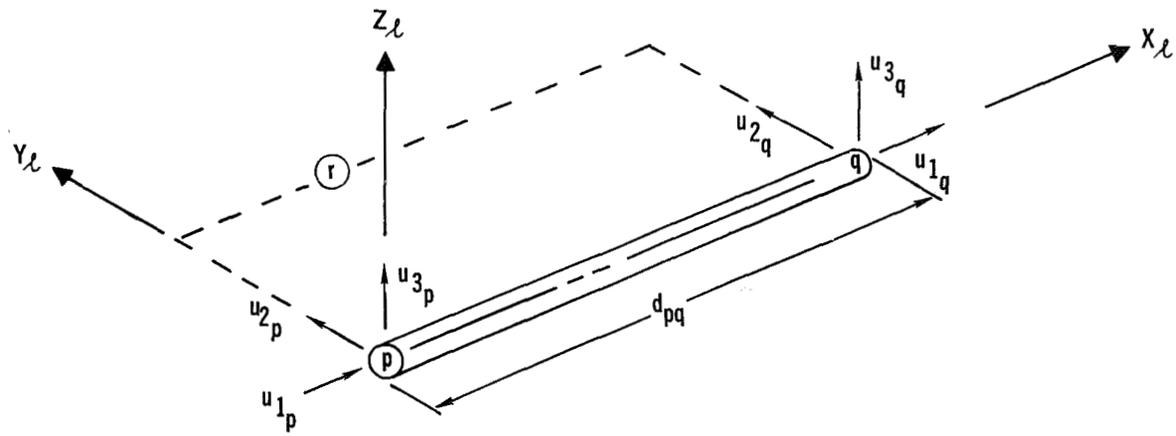
- (3) Elastic unloading through points 3, 4, and 5, followed by a continued decrease in stroke until the strut goes into tension. At this point, the strut load is governed by its tension load-stroke characteristics.

If at point 2 the stroke had not reversed direction, the load would remain constant until point 6 was reached. The force would then increase linearly with stroke and continue to follow the load-stroke curve until unloading took place. When the compressive stroke exceeds the maximum allowable stroke (SCMAX) the strut is assumed to have bottomed out.

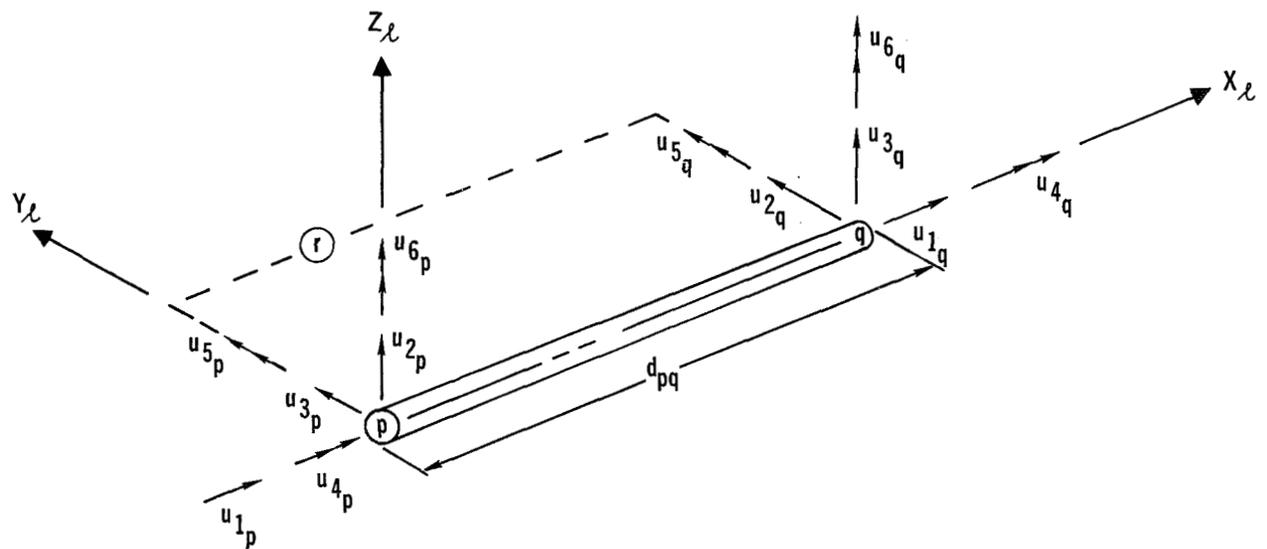
Two options are available for selecting the slope of the strut's elastic unloading characteristic. This slope may be either the slope of the next elastic portion of the load-stroke curve, or it may be some input value. The first option is governed by the assumption that at any point in the stroking of a strut, the elastic slope of the load-stroke curve is determined by the elastic properties of the uncrushed cartridges acting in series. Thus, when a cartridge crushes, its elastic characteristic is no longer reflected in the strut load-stroke curve.

For a bending strut which may or may not contain honeycomb attenuation, the bending properties (modulus of elasticity for bending displacements and the moment of inertia) are assumed to remain constant.

4.1.2.3 Coordinate Systems - Two types of right-hand orthogonal coordinate systems employed in the Landing Gear Option are the Local Coordinate System and the Global Coordinate System. Each axial strut or bending strut of either gear has its set of local coordinates as illustrated in Figure 4-19. The Local Coordinate System origin for each strut is located at node "p," with the X_ℓ axis aligned along the member axis. Positive X_ℓ is on the side of node "p" towards node "q." The Y_ℓ axis is perpendicular to X_ℓ and is located in the "pqr" plane. Positive Y_ℓ is on the side of X_ℓ towards "r." The Z_ℓ axis is then established using the right hand rule. Nodes p, q, and r, which establish each strut's local axis system, were preselected for all struts of the inverted tripod and cantilever gears.



(a) Axial Struts



(b) Bending Struts

FIGURE 4-19 STRUT DISPLACEMENTS IN LOCAL COORDINATE SYSTEM

Unless noted otherwise, in the following the terms "displacement" and "load" refer to incremental displacement or incremental load occurring during a step of the modified incremental stiffness method. The symbol " Δ " signifying an incremental quantity (see Equation (4-27)) is dropped for convenience.

Displacement notation of a typical axial strut capable of carrying axial load only is indicated in Figure 4-19(a) for the Local Coordinate System. Subscript 1 refers to displacement due to axial load while subscripts 2 and 3 are for displacements due to shear loads. Although axial struts can carry axial load only, displacements due to shear loads must be included in large displacement analysis as will be seen in the following sections. Cross-sectional area A , and modulus of elasticity E_A for axial displacements, must be specified for each axial strut relative to its Local Coordinate System. For an axial strut which contains honeycomb attenuation, this information is supplied in the form of the load-stroke curve shown in Figure 4-18.

Displacement notation of a typical bending strut capable of carrying axial load, shear in two directions, and bending in two directions is indicated in Figure 4-19(b) for the Local Coordinate System. Subscript 1 refers to a displacement due to axial load, subscripts 2 and 3 are for displacements due to shear loads, subscript 4 is for rotation due to torque, and subscripts 5 and 6 are for rotations due to moments. Although bending struts are not capable of carrying torsional loads, it is advantageous to include the torsional degrees of freedom since matrix transformation to a common coordinate system will be accomplished later and in this system all nodes will have six degrees of freedom. Cross-sectional area A , and modulus of elasticity E_A for axial displacements, must be specified for each bending strut relative to its Local Coordinate System. For a bending strut which contains honeycomb attenuation, this information is supplied in the form of the load-stroke curve shown in Figure 4-18. Moment of inertia I , and the modulus of elasticity E_B for bending displacements, are specified for each bending strut relative to its Local Coordinate System. The moment of inertia is assumed to be constant about any axis normal to X_ℓ .

Two Global Coordinate Systems employed in the Landing Gear Option are the Lander Coordinate System and the Surface Coordinate System as shown in

Figure 4-20. The origins of these coordinate systems are assumed coincident. Coordinates of all nodes of the gear are input in the Lander Coordinate System.

The Surface Coordinate System must be chosen such that the axis normal to the surface, X_S in Figure 4-20, is pointed outward. Three Euler angles defined as yaw (ψ), pitch (θ), and roll (ϕ), are used to orient the Surface Coordinate System with respect to the Lander Coordinate System. That is, the orientation of the Surface Coordinate System can be found by rotations in the order ψ , θ , and ϕ about the rotated Z_S , Y_S and X_S surface axes, respectively. The transformation matrix $[T_G]$ relating vector components in the Surface Coordinate System (VX_S, VY_S, VZ_S) to vector components in the Lander Coordinate System (VX_L, VY_L, VZ_L) is defined by Equation (4-28).

$$\begin{pmatrix} VX_S \\ VY_S \\ VZ_S \end{pmatrix} = [T_G] \begin{pmatrix} VX_L \\ VY_L \\ VZ_L \end{pmatrix} \quad (4-28)$$

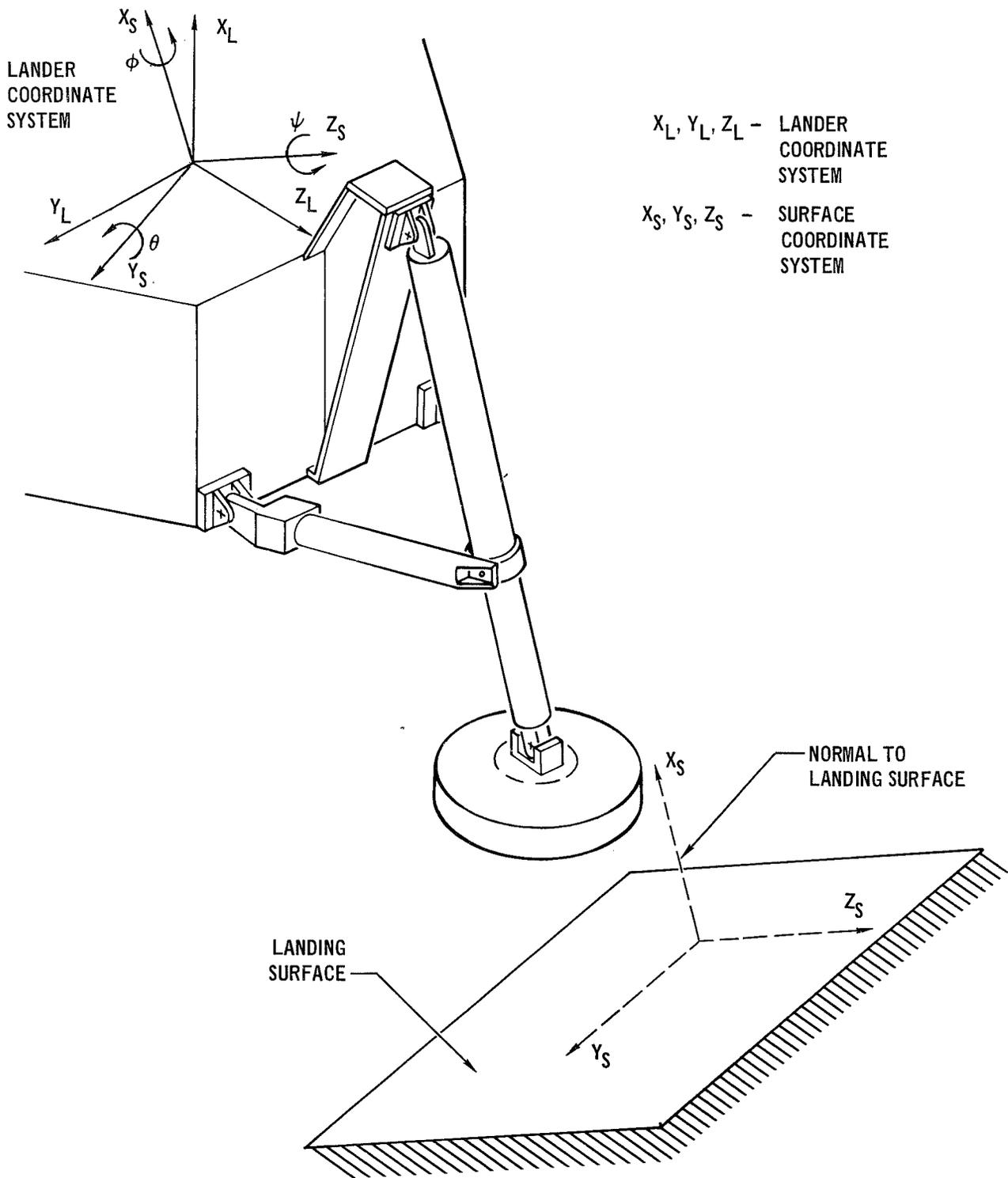
Terms in the transformation matrix $[T_G]$ are defined in Equation (4-29).

$$[T_G] = \begin{bmatrix} D_{11} & D_{12} & D_{13} \\ D_{21} & D_{22} & D_{23} \\ D_{31} & D_{32} & D_{33} \end{bmatrix} \quad (4-29)$$

$$\begin{aligned} D_{11} &= \cos \theta \cos \psi \\ D_{12} &= \cos \theta \sin \psi \\ D_{13} &= -\sin \theta \\ D_{21} &= \sin \phi \sin \theta \cos \psi - \cos \phi \sin \psi \\ D_{22} &= \sin \phi \sin \theta \sin \psi + \cos \phi \cos \psi \\ D_{23} &= \sin \phi \cos \theta \\ D_{31} &= \cos \phi \sin \theta \cos \psi + \sin \phi \sin \psi \\ D_{32} &= \cos \phi \sin \theta \sin \psi - \sin \phi \cos \psi \\ D_{33} &= \cos \phi \cos \theta \end{aligned}$$

Coordinates of all node points of the gear are transformed to surface coordinate orientation using Equation (4-28).

The Surface Coordinate System is established as the common system for all elements of a gear so that element total and incremental loads and displacements may be related to a common frame of reference. The term "total", when applied to loads or displacements, refers to the sum of incremental quantities which have occurred during the incremental steps which have been taken to the



X_L, Y_L, Z_L - LANDER
COORDINATE
SYSTEM

X_S, Y_S, Z_S - SURFACE
COORDINATE
SYSTEM

NOTE: EULER ANGLE ROTATIONS MUST BE CARRIED OUT IN THE ORDER OF YAW (ψ),
PITCH (θ), AND ROLL (ϕ) ABOUT SURFACE COORDINATE AXES.

FIGURE 4-20 LANDING GEAR COORDINATE SYSTEMS

current point of interest. Displacement notation for the axial strut and bending strut in the Surface Coordinate System is shown in Figure 4-21. In this figure and in the following discussion, the Surface Coordinate System X_S, Y_S, Z_S will be referred to as $\bar{X}, \bar{Y}, \bar{Z}$. Footpad loading conditions (see Section 4.1.2.7) are specified in this system. Matrix assembly and solution of equations for each step of the modified incremental stiffness method (see Section 4.1.2.6) are also accomplished in the surface system. That is, external loads and displacements at all nodes of the gear, as well as the total energy absorbed by the gear, are determined in surface coordinate components.

4.1.2.4 Incremental Stiffness Matrix - The incremental stiffness matrix, employed in the modified incremental stiffness method, is generated for both an "axial" strut and "bending" strut in the Local Coordinate System. This stiffness matrix will be shown to be the sum of the standard small displacement stiffness and the large displacement geometric stiffness. The geometric stiffness matrix is derived in Reference (8) using basic nonlinear theory in conjunction with the incremental stiffness method.

For each strut of a landing gear, incremental loads and displacements in the Local Coordinate System are related in matrix form by

$$\Delta\{F_\ell\} = [S_\ell] \Delta\{\delta_\ell\} \quad (4-30)$$

In Equation (4-30), $[S_\ell]$ is the incremental stiffness matrix; $\Delta\{F_\ell\}$ is a column matrix of applied incremental loads at the nodes; and $\Delta\{\delta_\ell\}$ is the column matrix of incremental displacements at the nodes. In the following, for convenience, the Δ symbol will be dropped. Accordingly, unless otherwise noted, the terms displacement and load refer to incremental quantities.

A. Axial Struts - As defined in Section 4.1.2.2, all elements of the inverted tripod gear and elements 1 and 3 of the cantilever gear (Figure 4-17(b)) are "axial" struts with load-carrying capability in the axial direction only. Displacement notation in the Local Coordinate System for a typical axial strut was shown in Figure 4-19(a).

The incremental stiffness matrix $[S_\ell]$ of an axial strut (in local coordinates) is the sum of the small displacement stiffness matrix shown in Figure 4-22 and the geometric stiffness matrix shown in Figure 4-23. The small displacement stiffness matrix of an axial strut is equivalent to the stiffness

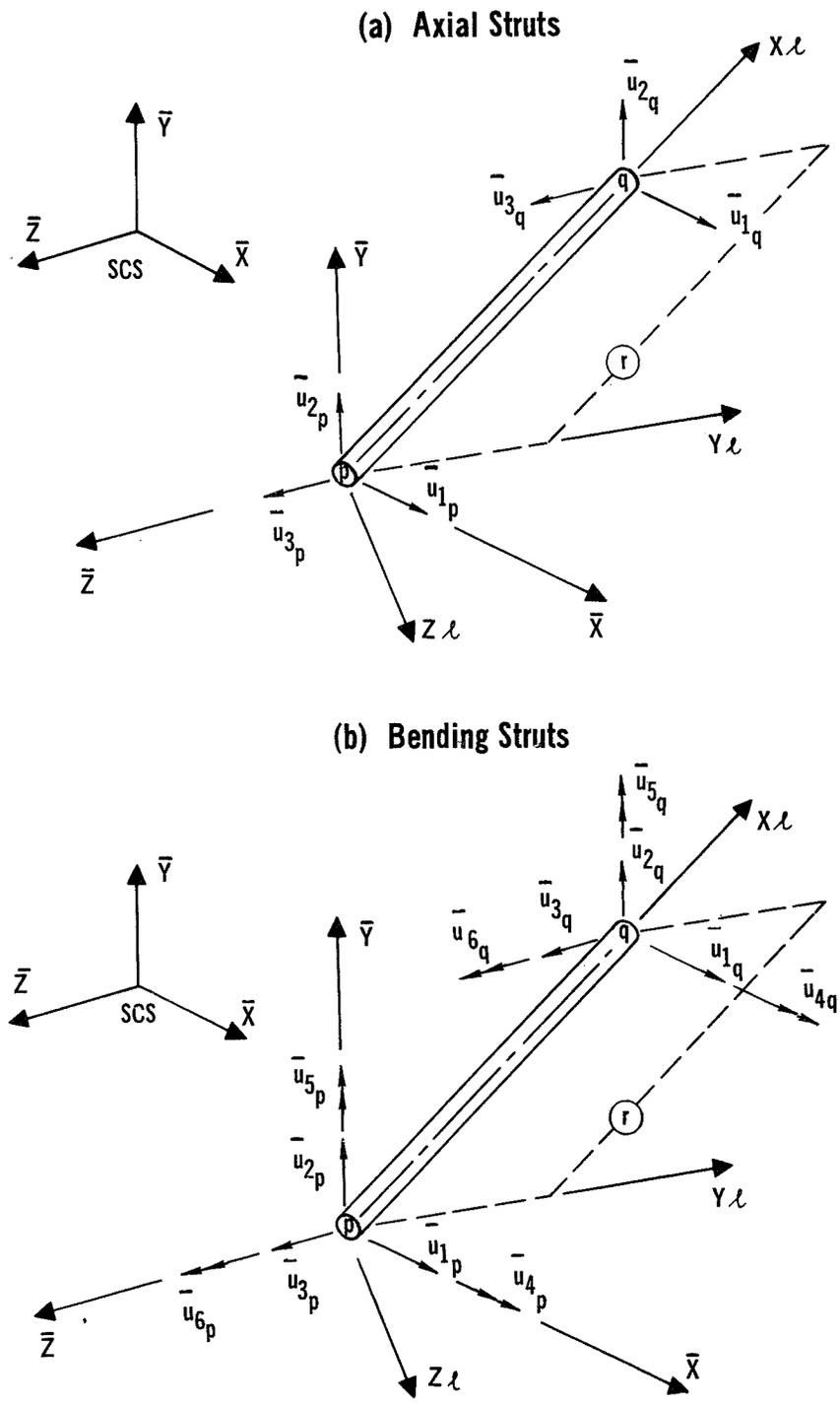


FIGURE 4-21 STRUT DISPLACEMENTS IN SURFACE COORDINATE SYSTEM

$$\begin{matrix}
 & u_{1p} & u_{2p} & u_{3p} & u_{1q} & u_{2q} & u_{3q} \\
 F_{1p} & \begin{bmatrix} \frac{AE_A}{d_{pq}} & & & \frac{-AE_A}{d_{pq}} & & & \\ & & & & & & \\ & & & & & & \\ \frac{-AE_A}{d_{pq}} & & & \frac{AE_A}{d_{pq}} & & & \\ & & & & & & \\ & & & & & & \\ F_{1q} & & & & & & \\ F_{2q} & & & & & & \\ F_{3q} & & & & & &
 \end{bmatrix}
 \end{matrix}$$

FIGURE 4-22 SMALL DISPLACEMENT STIFFNESS MATRIX OF AN AXIAL STRUT

$$\begin{array}{c}
 F_{1p} \\
 F_{2p} \\
 F_{3p} \\
 F_{1q} \\
 F_{2q} \\
 F_{3q}
 \end{array}
 \begin{bmatrix}
 u_{1p} & u_{2p} & u_{3p} & u_{1q} & u_{2q} & u_{3q} \\
 & \frac{P_0}{d_{pq}} & & & \frac{-P_0}{d_{pq}} & \\
 & & \frac{P_0}{d_{pq}} & & & \frac{-P_0}{d_{pq}} \\
 & & & & & \\
 & \frac{-P_0}{d_{pq}} & & & \frac{P_0}{d_{pq}} & \\
 & & \frac{-P_0}{d_{pq}} & & & \frac{P_0}{d_{pq}}
 \end{bmatrix}$$

FIGURE 4-23 GEOMETRIC STIFFNESS MATRIX OF AN AXIAL STRUT

matrix of the general bar element in the Center Body Option (see Figure 4-4) with $I_N = I_T = J = 0$ and only the translational degrees of freedom included. In Figure 4-22, A is the cross-sectional area; E_A the modulus of elasticity for axial displacements; and dpq the current length of the strut. For an axial strut composed of honeycomb attenuation, the terms $(A E_A / dpq)$ in the small displacement stiffness matrix are replaced by the slope of the load-stroke curve (see Figure 4-18) corresponding to the current axial stroke of the strut.

The geometric stiffness matrix, shown in Figure 4-23, includes the effect of "total" internal axial loads on the equilibrium equations in the presence of large rotational displacements. In the stiffness terms, P_0 is the total internal axial load in the strut corresponding to the current total axial displacement (stroke) of the strut. Positive values for P_0 signify that the strut is in tension.

B. Bending Struts - As defined in Section 4.1.2.2, the main strut of the cantilever gear (elements 2 and 4 in Figure 4-17(b)) is composed of "bending" struts with bending capability as well as axial capability. Displacement notation in the Local Coordinate System for a typical bending strut was shown in Figure 4-19(b).

The incremental stiffness matrix $[S_\rho]$ of a bending strut (in local coordinates) is the sum of the small displacement stiffness matrix shown in Figure 4-24 and the geometric stiffness matrix shown in Figure 4-25. The small displacement stiffness matrix is equivalent to the stiffness matrix of the general bar element in the Center Body Option (see Figure 4-4) with $J = 0$ and $I_N = I_T = I$. In Figure 4-24, A is the cross-sectional area; E_A the modulus of elasticity for axial displacements; E_B the modulus of elasticity for bending displacements; I the moment of inertia about any axis normal to the strut; and dpq the current length of the strut. For a bending strut whose axial capability is defined by honeycomb attenuation, the terms $(A E_A / dpq)$ in the small displacement stiffness matrix are replaced by the slope of the load-stroke curve (see Figure 4-18) corresponding to the current axial stroke of the strut.

The geometric stiffness matrix, shown in Figure 4-25, includes the effect of total internal axial loads on the equilibrium equations in the presence of large rotational displacements. In Figure 4-25, P_0 is the total internal axial load in the strut corresponding to the current total axial displacement (stroke). Positive values for P_0 signify tension.

	u_{1p}	u_{2p}	u_{3p}	u_{1q}	u_{2q}	u_{3q}	u_{4p}	u_{5p}	u_{6p}	u_{4q}	u_{5q}	u_{6q}
F_{1p}	$\frac{AE_A}{d_{pq}}$			$-\frac{AE_A}{d_{pq}}$								
F_{2p}		$\frac{12 E_B l}{(d_{pq})^3}$			$-\frac{12 E_B l}{(d_{pq})^3}$				$\frac{6 E_B l}{(d_{pq})^2}$			$\frac{6 E_B l}{(d_{pq})^2}$
F_{3p}			$\frac{12 E_B l}{(d_{pq})^3}$			$-\frac{12 E_B l}{(d_{pq})^3}$		$-\frac{6 E_B l}{(d_{pq})^2}$			$-\frac{6 E_B l}{(d_{pq})^2}$	
F_{1q}	$-\frac{A E_A}{d_{pq}}$			$\frac{A E_A}{d_{pq}}$								
F_{2q}		$-\frac{12 E_B l}{(d_{pq})^3}$			$\frac{12 E_B l}{(d_{pq})^3}$				$-\frac{6 E_B l}{(d_{pq})^2}$			$-\frac{6 E_B l}{(d_{pq})^2}$
F_{3q}			$-\frac{12 E_B l}{(d_{pq})^3}$			$\frac{12 E_B l}{(d_{pq})^3}$		$\frac{6 E_B l}{(d_{pq})^2}$			$\frac{6 E_B l}{(d_{pq})^2}$	
M_{4p}												
M_{5p}			$-\frac{6 E_B l}{(d_{pq})^2}$			$\frac{6 E_B l}{(d_{pq})^2}$		$\frac{4 E_B l}{d_{pq}}$			$\frac{2 E_B l}{d_{pq}}$	
M_{6p}		$\frac{6 E_B l}{(d_{pq})^2}$			$-\frac{6 E_B l}{(d_{pq})^2}$			$\frac{4 E_B l}{d_{pq}}$			$\frac{2 E_B l}{d_{pq}}$	
M_{4q}												
M_{5q}			$-\frac{6 E_B l}{(d_{pq})^2}$			$\frac{6 E_B l}{(d_{pq})^2}$		$\frac{2 E_B l}{d_{pq}}$			$\frac{4 E_B l}{d_{pq}}$	
M_{6q}		$\frac{6 E_B l}{(d_{pq})^2}$			$-\frac{6 E_B l}{(d_{pq})^2}$			$\frac{2 E_B l}{d_{pq}}$			$\frac{4 E_B l}{d_{pq}}$	

FIGURE 4-24 SMALL DISPLACEMENT STIFFNESS MATRIX OF A BENDING STRUT

	u_{1p}	u_{2p}	u_{3p}	u_{1q}	u_{2q}	u_{3q}	u_{4p}	u_{5p}	u_{6p}	u_{4q}	u_{5q}	u_{6q}
F_{1p}												
F_{2p}		$\frac{6 P_0}{5 d_{pq}}$			$\frac{-6 P_0}{5 d_{pq}}$				$\frac{P_0}{10}$			$\frac{P_0}{10}$
F_{3p}			$\frac{6 P_0}{5 d_{pq}}$			$\frac{-6 P_0}{5 d_{pq}}$		$\frac{-P_0}{10}$			$\frac{-P_0}{10}$	
F_{1q}												
F_{2q}		$\frac{-6 P_0}{5 d_{pq}}$			$\frac{6 P_0}{5 d_{pq}}$				$\frac{-P_0}{10}$			$\frac{-P_0}{10}$
F_{3q}			$\frac{-6 P_0}{5 d_{pq}}$			$\frac{6 P_0}{5 d_{pq}}$		$\frac{P_0}{10}$			$\frac{P_0}{10}$	
M_{4p}												
M_{5p}			$\frac{-P_0}{10}$			$\frac{P_0}{10}$		$\frac{2 P_0 d_{pq}}{15}$			$\frac{-P_0 d_{pq}}{30}$	
M_{6p}		$\frac{P_0}{10}$			$\frac{-P_0}{10}$				$\frac{2 P_0 d_{pq}}{15}$			$\frac{-P_0 d_{pq}}{30}$
M_{4q}												
M_{5q}			$\frac{-P_0}{10}$			$\frac{P_0}{10}$		$\frac{-P_0 d_{pq}}{30}$			$\frac{2 P_0 d_{pq}}{15}$	
M_{6q}		$\frac{P_0}{10}$			$\frac{-P_0}{10}$				$\frac{-P_0 d_{pq}}{30}$			$\frac{2 P_0 d_{pq}}{15}$

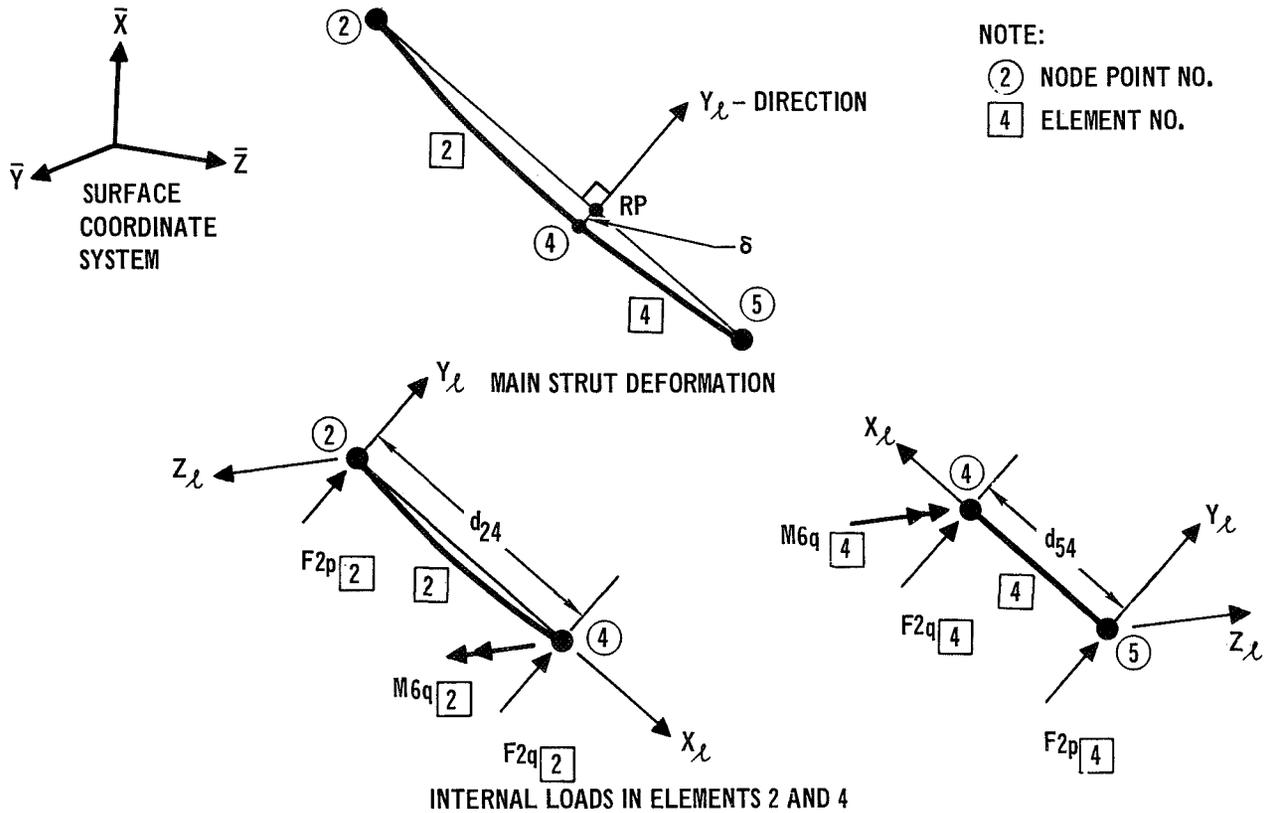
FIGURE 4-25 GEOMETRIC STIFFNESS MATRIX OF A BENDING STRUT

4.1.2.5 Internal Loads - For each iteration at each step of the modified incremental stiffness method, the total internal loads in each strut must be determined for the current displaced state of the strut. The total internal loads must be calculated so that load unbalances can be determined (see Section 4.1.2.1).

For axial struts, the total internal loads are simply the axial tension or compression forces which would have to be applied to the ends of the strut to extend or compress it to its present state. The extension or compression of a strut (the stroke) is the difference between the current length and the original length. Total axial force is the product of stroke and the small displacement stiffness term ($A E_A / dpq$) defined in Section 4.1.2.4. A positive stroke corresponds to extension of the strut and yields a positive (tensile) total axial force. For an axial strut composed of honeycomb attenuation, the total axial force is determined by the load-stroke curve (see Figure 4-18).

The total internal loads for bending struts consist of axial forces, shear forces, and bending moments. The axial forces are calculated in the same way as for axial struts. Calculation of bending loads (shear forces and bending moments) in the deflected main strut of the cantilever gear (elements 2 and 4 in Figure 4-17) is shown in Figure 4-26. Because the main strut is pinned at both ends, loads causing bending are shear loads ($F2p_{[2]}$ and $F2p_{[4]}$) applied at the ends in the plane of the deformed strut and directed along the Y_ℓ axis. The Y_ℓ direction is defined by a line passing through node 4 perpendicular to the line connecting nodes 2 and 5 and intersecting it at reference point RP. Since there is bending continuity at node 4, internal loads in elements 2 and 4 are shears ($F2q_{[2]}$ and $F2q_{[4]}$) and equal bending moments ($M6q_{[2]}$ and $M6q_{[4]}$). Coordinates of the floating reference point (RP) for elements 2 and 4 are also calculated in Figure 4-26.

4.1.2.6 Matrix Assembly and Solution of Equations - As explained in Section 4.1.2.3, the Surface Coordinate System is used for all elements of a gear to relate total and incremental loads and displacements to a common frame of reference. For each element of the gear, the incremental stiffness matrix is transformed from Local to Surface Coordinate System as described in Section 4.1.1.2.



DEFINITIONS:

$$\begin{aligned}
 a &= \bar{X}_5 - \bar{X}_2 & d_{25} &= d_{24} + d_{54} \\
 b &= \bar{Y}_5 - \bar{Y}_2 & E_{B2} &= \text{BENDING MODULUS OF ELASTICITY (ELEMENT 2)} \\
 c &= \bar{Z}_5 - \bar{Z}_2 & E_{B4} &= \text{BENDING MODULUS OF ELASTICITY (ELEMENT 4)} \\
 e &= \bar{X}_5 - \bar{X}_4 & I_2 &= \text{MOMENT OF INERTIA (ELEMENT 2)} \\
 f &= \bar{Y}_5 - \bar{Y}_4 & I_4 &= \text{MOMENT OF INERTIA (ELEMENT 4)} \\
 g &= \bar{Z}_5 - \bar{Z}_4
 \end{aligned}$$

$$\delta = \frac{1}{d_{25}} \sqrt{(bg - cf)^2 + (ce - ag)^2 + (af - be)^2}$$

$$T = \frac{\delta}{\left[\frac{(d_{24})^3 d_{54}}{3} - \frac{(d_{24})^4 d_{54}}{3 d_{25}} \right] / E_{B2} I_2 + \left[\frac{(d_{24})^2 (d_{25})^2}{3} - (d_{24})^3 d_{25} + (d_{24})^4 - \frac{(d_{24})^5}{3 d_{25}} \right] / E_{B4} I_4}$$

REFERENCE POINT COORDINATES:

$$\begin{aligned}
 \bar{X}_{RP} &= \bar{X}_4 + (c^2 e - acg - abf + b^2 e) / (d_{25})^2 \\
 \bar{Y}_{RP} &= \bar{Y}_4 + (a^2 f - abe - bcf + c^2 f) / (d_{25})^2 \\
 \bar{Z}_{RP} &= \bar{Z}_4 + (b^2 g - bcf - ace + a^2 g) / (d_{25})^2
 \end{aligned}$$

TOTAL INTERNAL LOADS:

$$\begin{aligned}
 F_{2p[2]} &= d_{54} T = -F_{2q[2]} & M_{6q[2]} &= M_{6q[4]} = d_{24} d_{54} T \\
 F_{2p[4]} &= d_{24} T = -F_{2q[4]}
 \end{aligned}$$

FIGURE 4-26 INTERNAL BENDING LOADS IN MAIN STRUT OF CANTILEVER GEAR

Before matrix assembly can be accomplished, the incremental stiffness matrix in the Surface Coordinate System for all elements of the gear must be of the same order. All elements of the inverted tripod gear are axial struts and hence have incremental stiffness matrices of the same order. For the cantilever gear, the incremental stiffness matrices of the axial struts (elements 1 and 3 in Figure 4-17) must be expanded to the row and column format of the bending strut stiffness matrices of elements 2 and 4. This is accomplished by adding rows and columns of zeros corresponding to the six rotational degrees of freedom.

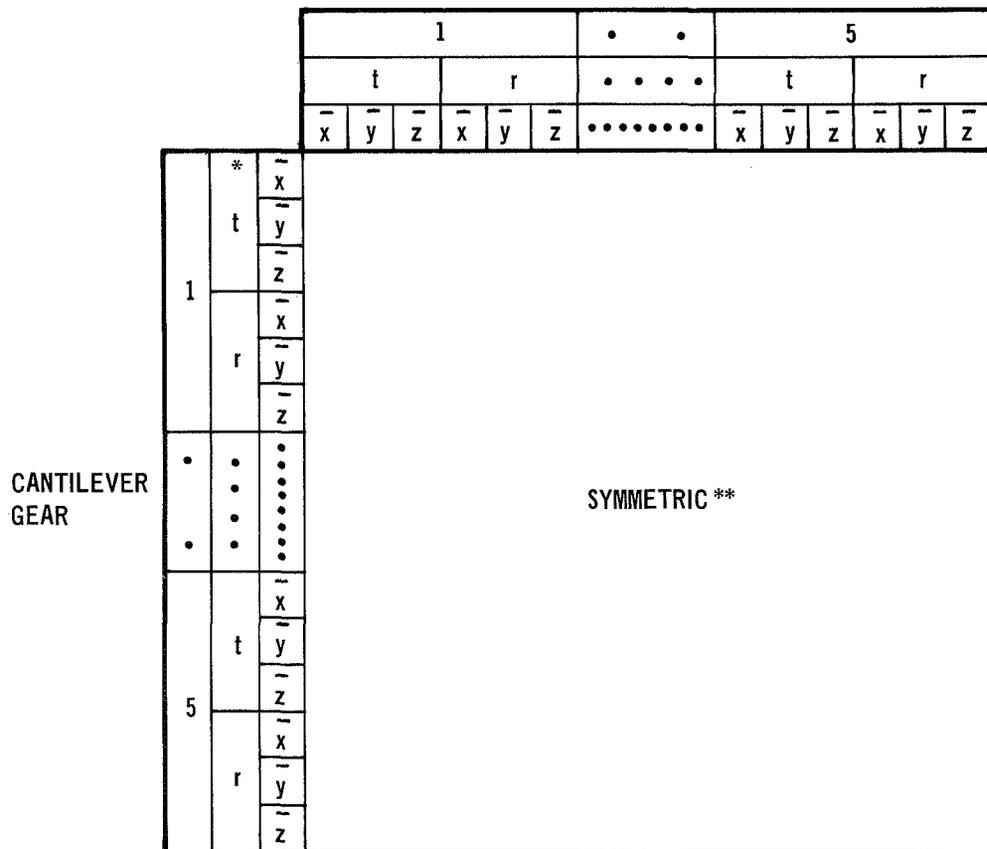
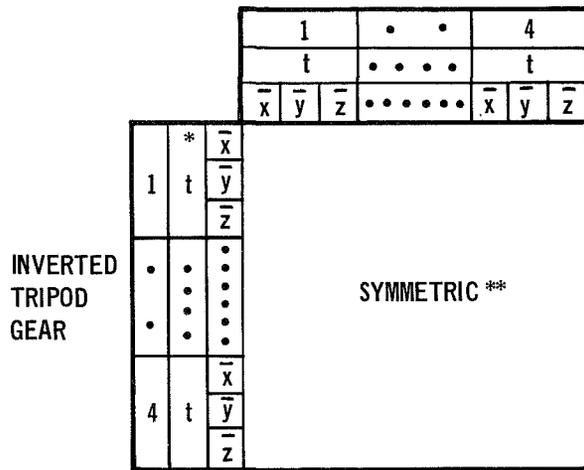
The total incremental stiffness matrix $[\bar{S}]$ (in the Surface Coordinate System) for the assembled gear is generated by systematically adding the transformed element stiffness matrices (according to the p and q nodes of the element) as shown in Figure 4-27. Node points on the inverted tripod gear are numbered consecutively from 1 to 4 (see Figure 4-17). The incremental stiffness matrix $[\bar{S}]$ of this gear is assembled with a row and column format corresponding to the three translational degrees of freedom in the Surface Coordinate System at each node in sequence. In this case, the size of the stiffness matrix is 12 by 12. For the cantilever gear, node points are numbered consecutively from 1 to 5. The incremental stiffness matrix $[\bar{S}]$ of this gear is assembled with a row and column format corresponding to the three translational followed by the three rotational degrees of freedom in the Surface Coordinate System at each node in sequence. In this case, the size of the stiffness matrix is 30 by 30.

The assembled incremental stiffness matrix $[\bar{S}]$ is related to the column matrices of incremental surface coordinate loads and displacements at each node by Equation (4-31).

$$\Delta\{\bar{F}\} = [\bar{S}] \Delta\{\bar{\delta}\} \quad (4-31)$$

For the fixed idealization of the inverted tripod gear (see Figure 4-17), nodes 1, 2, and 3 are pinned and thus have zero translational displacements. Since only translational displacements are included in the stiffness formulation for this gear, the degrees of freedom of interest are the translations of node 4. Therefore, for the inverted tripod, Equation (4-31) reduces to

$$\Delta\{\bar{F}'\} = [\bar{S}'] \Delta\{\bar{\delta}'\} \quad (4-32)$$



* SYMBOLS t AND r INDICATE TRANSLATIONAL AND ROTATIONAL INCREMENTAL DISPLACEMENTS, RESPECTIVELY.

** TERMS IN THE MATRICES OF INDIVIDUAL ELEMENT INCREMENTAL STIFFNESSES IN THE SURFACE COORDINATE SYSTEM ARE SUCCESSIVELY ADDED INTO THE APPROPRIATE LOCATIONS IN THE FRAMEWORK OF THIS MATRIX.

FIGURE 4-27 ASSEMBLED INCREMENTAL STIFFNESS MATRIX FORMAT

In this equation, $\Delta\{\bar{F}'\}$ is a column matrix of three incremental force components at node 4; $\Delta\{\bar{\delta}'\}$ a column matrix of three incremental translations at node 4; and $[\bar{S}']$ a 3 by 3 matrix of stiffness terms, extracted from $[\bar{S}]$, corresponding to the degrees of freedom retained at node 4.

For the fixed idealization of the cantilever gear (see Figure 4-17), nodes 1, 2, and 3 are pinned and hence have zero translational displacements. Accordingly, equations associated with the 3 translational degrees of freedom at each of these nodes are not of interest. Elements 1 and 3 have zero bending capability. Therefore, the six equations associated with the rotational degrees of freedom at nodes 1 and 3 are not of interest. The remaining degrees of freedom for a cantilever gear are three rotations at node 2, three translations and three rotations at node 4, and three translations and three rotations at node 5. Thus, for the cantilever gear Equation (4-31) reduces to

$$\Delta\{\bar{F}''\} = [\bar{S}''] \Delta\{\bar{\delta}''\} \quad (4-33)$$

In Equation (4-33), $\Delta\{\bar{F}''\}$ is a column matrix containing three incremental moment components at node 2, three incremental force components and three incremental moment components at node 4, and three incremental force components and three incremental moment components at node 5; $\Delta\{\bar{\delta}''\}$ a column matrix of the fifteen incremental displacements described above; and $[\bar{S}'']$ a 15 by 15 matrix of stiffness terms, extracted from $[\bar{S}]$, corresponding to the 15 degrees of freedom retained.

Equation (4-32) or (4-33) is solved repeatedly at each step of the modified incremental stiffness method (see Section 4.1.2.1). A least squares solution technique is programmed for solution of these equations. The independent variables for each of these solutions are discussed in Section 4.1.2.7. After the incremental nodal displacements have been determined, the updated nodal locations can be found as the sum of the previous nodal coordinates and the incremental nodal displacements. For the updated nodal locations, the total internal loads can then be determined as described in Section 4.1.2.5.

4.1.2.7 Applied Displacement Options - For each step of the modified incremental stiffness method, the governing matrix equation, Equation (4-32) for an inverted tripod gear or Equation (4-33) for a cantilever gear, is solved as outlined below. Both of these equations are analogous to Equation

(4-27) and the solution technique is the same as discussed in Section 4.1.2.1.

- (i) Equation (4-32) (or Equation (4-33)) is solved once for the case where $\Delta \{\bar{\delta}'\}$ (or $\Delta \{\bar{\delta}''\}$), the total applied displacement vector divided by the number of incremental steps, is the (known) independent variable.
- (ii) Equation (4-32) (or Equation (4-33)) is solved repeatedly for the case where $\Delta \{\bar{F}'\}$ (or $\Delta \{\bar{F}''\}$), the load unbalance vector, is the (known) independent variable. The load unbalance vector is calculated as the difference between applied external nodal loads and total internal nodal loads. Calculation of internal loads is described in Section 4.1.2.5.

Two options are available for specifying the total applied displacement vector discussed in (i) above. In the first option, total displacement of the footpad joint (node 4 for the inverted tripod gear and node 5 for cantilever gear) normal to a landing surface, and the coefficient of friction of the surface are specified. For example, suppose it is desired to investigate the displaced state of the gear at each of N steps for an applied normal displacement value of D units. For this case, the program determines N equilibrium configurations for the gear. For each of these configurations, the footpad joint is located in one of N different planes spaced D/N units apart. Whether the footpad slides on the landing surface as the footpad joint is displaced from the (n-1)st plane to the nth plane ($n = 1, \dots, N$) depends on the friction coefficient. As explained in Section 4.1.2.3, the landing surface can be arbitrarily oriented with respect to the Lander Coordinate System.

In the second option available for specifying the applied displacement vector, three components of footpad joint displacement are specified in the Surface Coordinate System ($\bar{X}, \bar{Y}, \bar{Z}$). For example, if it is desired to investigate the displaced state of the gear at each of N steps when the applied footpad joint displacement components are $\bar{X}_f, \bar{Y}_f,$ and $\bar{Z}_f,$ the program determines N equilibrium configurations for the gear. In each of these configurations, the footpad joint is located at one of the positions

$(\bar{X}_o + n \bar{X}_f/N, \bar{Y}_o + n \bar{Y}_f/N, \bar{Z}_o + n \bar{Z}_f/N)$ where $n = 1, \dots, N$. The original position of the footpad joint (see Figure 4-17) is assumed to be $(\bar{X}_o, \bar{Y}_o, \bar{Z}_o)$.

4.1.2.8 Energy Absorption - For each step of the landing gear displacement history, energy absorbed by the gear during the step is calculated in both Surface and Lander Coordinate System components.

Although an element representing the footpad is not included in the landing gear idealization, energy associated with crushing of footpad honeycomb can be included in the analysis. Footpad honeycomb attenuation is idealized with up to three levels of crushing in compression as shown in Figure 4-28.

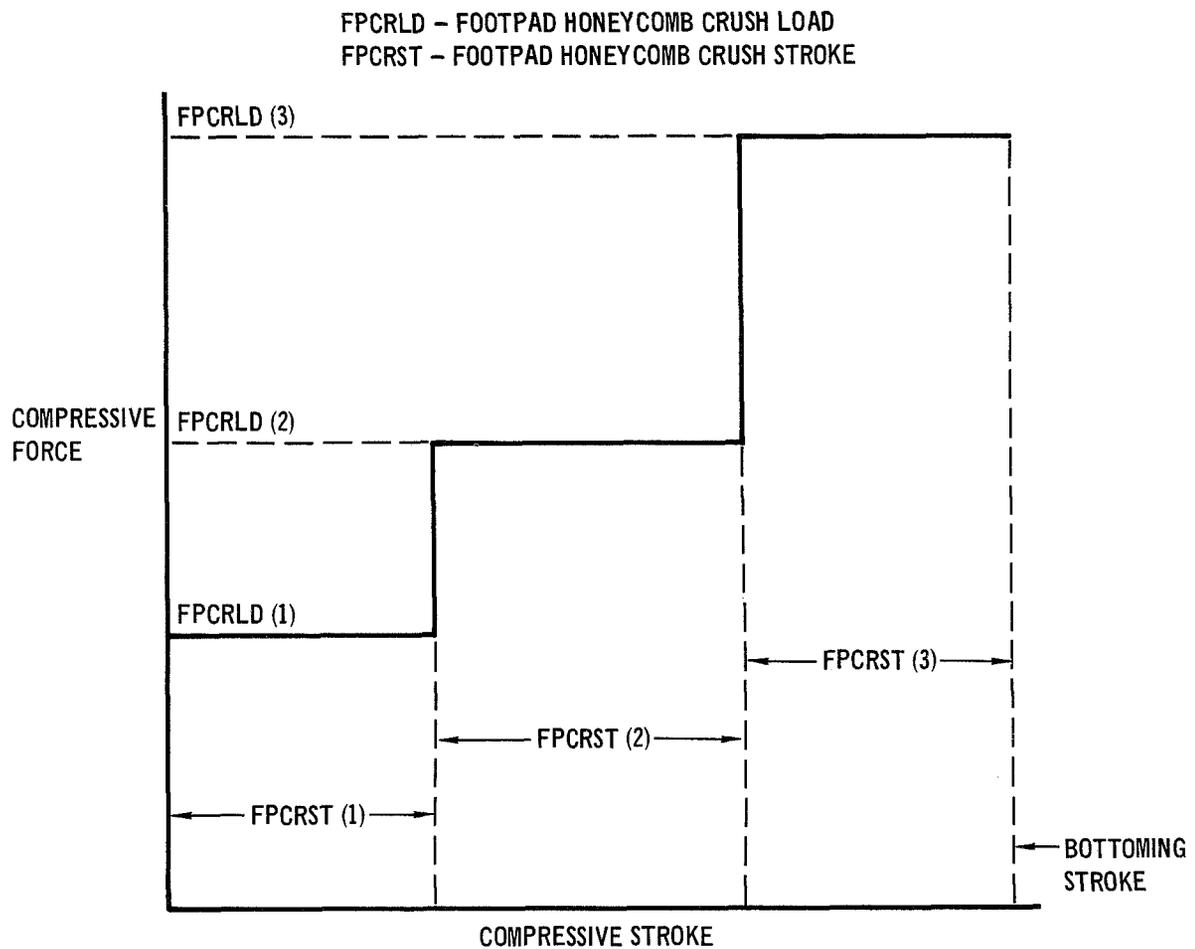


FIGURE 4-28 ASSUMED LOAD-STROKE CURVE FOR FOOTPAD HONEYCOMB

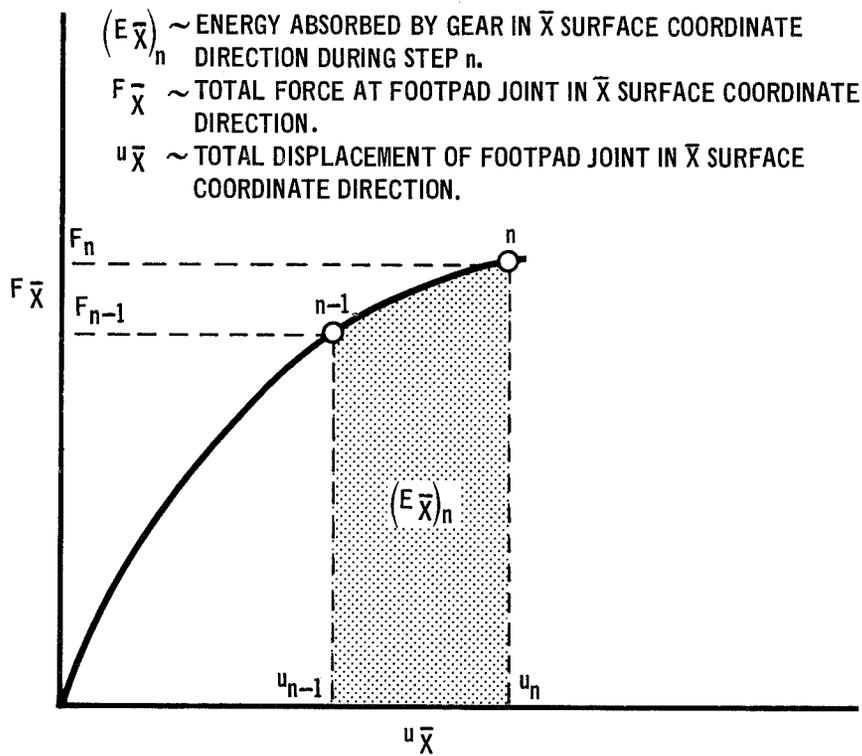
Each surface coordinate energy component absorbed at the footpad joint during a step is the sum of strut crush energy and footpad crush energy. Strut crush energy in a given surface coordinate direction is calculated as the product of the average force component (in that direction) over the step and the incremental displacement component (in the same direction) which occurred during the step. Figure 4-29 presents an example calculation of strut crush energy component absorbed in the \bar{X} surface coordinate direction during the n^{th} incremental step. For a gear which has footpad attenuation with up to three levels of crush force, footpad crush energy may be added into the surface coordinate energy component in the direction normal to the landing surface. For a given incremental step, footpad crush energy is added into this energy component if the force component in the normal direction (see Figure 4-20) at the footpad joint exceeds one of the footpad honeycomb crush forces (Figure 4-28) for the first time. When this occurs, the associated level of honeycomb crush is assumed to have bottomed out during the step. Footpad crush energy component associated with this crushing is the product of crush load (FPCRLD) and the crush stroke (FPCRST). For example, the surface coordinate direction normal to the landing surface is \bar{X} (Figure 4-20). If during incremental step n , the force in the \bar{X} direction at the footpad joint exceeds the first crush force FPCRLD(1) in Figure 4-28, and this has not occurred for any previous step, the energy term FPCRLD(1) times FPCRST(1) will be added to the \bar{X} strut crush energy component for this step.

Components of energy absorption during a given incremental step are obtained similarly in Lander Coordinate System components. In both coordinate systems, total energy absorption components are determined after n incremental steps as the sum of the energy components obtained in each of the n steps taken.

4.2 Program Description - Organization of the Structural Analysis Program is described in this section. Included are flow diagrams and discussions of subroutine functions for both the Center Body Option and Landing Gear Option.

4.2.1 Subroutines - The Structural Analysis Program is divided into nine Fortran OVERLAY segments. OVERLAY organization and description of the function of each subroutine are shown in Figure 4-30. This organization is required to stay within the allotted core storage requirements. The main OVERLAY (0, 0)

acts as the principal executive routine and determines the order in which primary OVERLAYS (1, 0) through (5, 0) are called. OVERLAY (1, 0) processes the data set header cards and handles the data sets and initialization relating to the Center Body Option. OVERLAY (2, 0) generates the total stiffness matrix for the Center Body Option. OVERLAY (3, 0) determines the displacements and rotations for the Center Body Option and prints the results. The modal analysis portion of the Center Body Option is handled in OVERLAY (4, 0). OVERLAY (5, 0) contains the routines for the Landing Gear Option and has three supporting secondary overlays; OVERLAY (5, 1) which reads the landing gear data cases, OVERLAY (5, 2) which contains the executive routine for the inverted tripod gear, and OVERLAY (5, 3) which contains the executive routine for the cantilever gear.



FOR STEP n :

$$\begin{aligned}
 (E_{\bar{x}})_n &= \text{AREA UNDER CURVE BETWEEN POINTS } n-1 \text{ AND } n \\
 &= \frac{(F_n + F_{n-1})}{2} (u_n - u_{n-1})
 \end{aligned}$$

FIGURE 4-29 EXAMPLE CALCULATION OF STRUT CRUSH ENERGY

OVERLAY	SUBROUTINE	FUNCTION
(0, 0)	MAIN ERPNT1 ERPNT2 WRSTRK	PROGRAM EXECUTIVE ROUTINE ERROR PROCESSING ROUTINE FOR ALL OVERLAYS SECOND ENTRY POINT TO ERPNT1 SPARSE MATRIX PRINT ROUTINE
(1, 0)	INITAL DATSET RDDATA SHRPAN CROSSX	EXECUTIVE ROUTINE HEADER CARD PROCESSOR AND CENTER BODY OPTION DATA CASE READER CENTER BODY OPTION DATA CASE PROCESSOR CONVERT SHEAR PANELS TO EQUIVALENT DIAGONAL BAR ELEMENTS CALCULATE THE CROSS PRODUCT OR DOT PRODUCT OF TWO VECTORS
(2, 0)	STIFF SETSTF STFTRN STORMS TRASMK WRBDAT BRSTRA WRSTDK	EXECUTIVE ROUTINE INITIALIZE STRUCTURAL STIFFNESS MATRIX STORAGE ARRAY IN A SPARSE BLOCKED FORMAT CALCULATE ELEMENT STIFFNESS AND TRANSFORMATION MATRICES CONSTRUCT ASSEMBLED STIFFNESS MATRIX TRANSFORM ELEMENT STIFFNESS MATRICES FROM LOCAL TO GLOBAL COORDINATE SYSTEM PRINT ELEMENT STIFFNESS AND TRANSFORMATION MATRICES SECOND ENTRY POINT TO WRBDAT, PRINTS TRANSFORMED ELEMENT STIFFNESS MATRICES WRITE ELEMENT STIFFNESS AND TRANSFORMATION MATRICES ON FILE 2
(3, 0)	FINIAL PANDTK SOLVE PNTFMV FMBARS	EXECUTIVE ROUTINE PRINT AND STORE ON FILE 9 THE ASSEMBLED STIFFNESS MATRIX SET-UP FOR SOLUTION DISPLACEMENTS AND ROTATIONS DETERMINED USING ITERATIVE METHOD. GLOBAL FORCES AND MOMENTS ON JOINTS CALCULATED PRINT DISPLACEMENTS, ROTATIONS, AND GLOBAL FORCES AND MOMENTS ELEMENT BAR FORCES AND MOMENTS CALCULATED USING RESULTS OF SOLVE
(4, 0)	NLMDAL PUNPACK REDUCE STFMAS FNALEV EEPNT TRIDIA EIGVAL EIGVEC PRNT1 PRNT2 PIA721	EXECUTIVE ROUTINE DATA HANDLING ROUTINE FOR SUBROUTINE REDUCE PRINT ASSEMBLED STIFFNESS MATRIX AND REDUCE TO DESIRED SIZE (REDUCED MATRIX 102 x 102 OR LESS) CREATES MATRIX SYSTEM, USING INPUT DIAGONAL MASS MATRIX AND REDUCED STIFFNESS MATRIX, FOR WHICH EIGENVALUES CAN BE FOUND EIGENVECTORS OF REDUCED SYSTEM TRANSFORMED INTO EIGENVECTORS OF FULL SYSTEM WRITE EIGENVALUES AND EIGENVECTORS MATRIX SYSTEM IS TRI-DIAGONALIZED USING HOUSEHOLDER'S METHOD SPECIFIED NUMBER OF SMALLEST EIGENVALUES OF MATRIX SYSTEM ARE CALCULATED USING ORTEGA'S METHOD (PLUS SIX RIGID BODY NATURAL FREQUENCIES OF ZERO) CALCULATES EIGENVECTORS ASSOCIATED WITH LOWEST EIGENVALUES USING WILKINSON'S METHOD MODAL DATA OUTPUT ROUTINE FOR REDUCED SYSTEM SECOND ENTRY POINT OF PRNT1 TO OUTPUT MODAL DATA OF FULL SYSTEM PRINT SUPPORT ROUTINE FOR PRNT1 AND PRNT2
(5, 0)	GEAREX OUTPUT ENERGY TRNFSM TRALMG STFMIT TRANSM DMFSS DMLSS STRUT	EXECUTIVE ROUTINE LANDING GEAR OPTION OUTPUT COMPUTE AND SUM ENERGY QUANTITIES TRANSFORM ELEMENT STIFFNESS MATRIX TRANSFORM FORCE-MOMENTS FROM LOCAL TO GLOBAL COORDINATES COMPUTE NON-BENDING ELEMENT STIFFNESS MATRIX COMPUTE ELEMENT LENGTH AND TRANSFORMATION MATRIX DETERMINE THE RANK OF A SYMMETRIC POSITIVE SEMI-DEFINITE MATRIX CALCULATE THE LEAST SQUARES SOLUTION OF A SYSTEM OF SIMULTANEOUS EQUATIONS WITH SYMMETRIC POSITIVE SEMI-DEFINITE COEFFICIENT MATRIX WHOSE RANK IS KNOWN COMPUTE ELEMENT AXIAL LOAD AND STIFFNESS
(5, 1)	INPUT GEOM	READ LANDING GEAR OPTION DATA CASES COMPUTE LANDER COORDINATE TO SURFACE COORDINATE TRANSFORMATION MATRIX FROM EULER ANGLES
(5, 2)	INVTRP	EXECUTE INVERTED TRIPOD DATA COMMANDS
(5, 3)	CANTIL BNDLDS STFMCF	EXECUTE CANTILEVER DATA COMMANDS COMPUTE BENDING LOADS FOR CANTILEVER GEAR COMPUTE BENDING ELEMENT STIFFNESS MATRIX

FIGURE 4-30 STRUCTURAL ANALYSIS PROGRAM SUBROUTINES

4.2.2 Flow Diagram - The Structural Analysis Program is shown schematically in Figure 4-31. While detailed steps, such as those required in the iteration loop of the Center Body Option, are not presented, the basic sequence of events is shown for both options of the program. A listing of the program is given in Appendix H.

4.3 Program Operation - Information necessary to operate both options of the Structural Analysis Program is contained in this section.

4.3.1 Center Body Option - This section includes definition of input requirements and format, and output interpretation for the Center Body Option of the program. Examples of input and output data for a typical problem are contained in Appendices A and B.

4.3.1.1 Input Data - For each Center Body Option data set, a data case header card and information describing geometry, supports, applied loads and displacements, member properties, and indicators needed to control program operation are required as input data. Multiple data cases may be run by stacking data sets. Each data set must consist of the following:

Header Card

Data Cards

Data Option Indicator (NAMELIST) Card(s)

A. Header Card - The first card of a data set must be a header card. This card indicates that the Center Body Option will be employed and must contain the characters STRUCTURE in columns 1 through 9 as indicated in Figure 4-32.

B. Data Cards - The input format for data cards employs a system of code numbers, located in column 1, to identify the type of data being read. The nine code numbers and corresponding type of information being input are as follows:

- (0) Comment cards
- (1) Joint information cards
- (2) Reference point information cards
- (3) Force and moment limits or specified displacement and rotation cards
- (4) Specified force vector cards
- (5) Specified moment vector cards
- (6) Bar information cards
- (7) Rectangular shear web information cards
- (8) Data terminator card

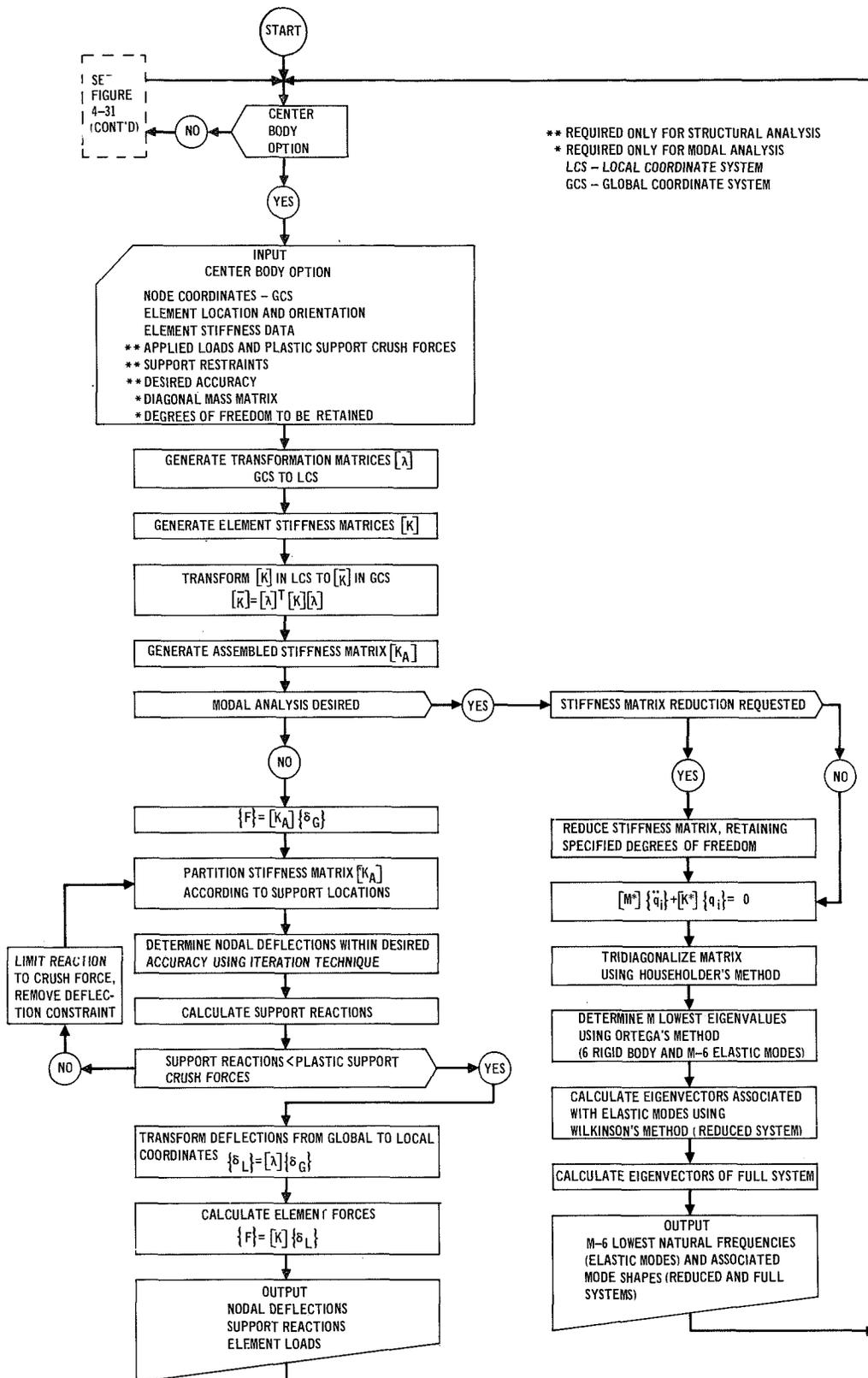


FIGURE 4-31 FLOW DIAGRAM STRUCTURAL ANALYSIS PROGRAM

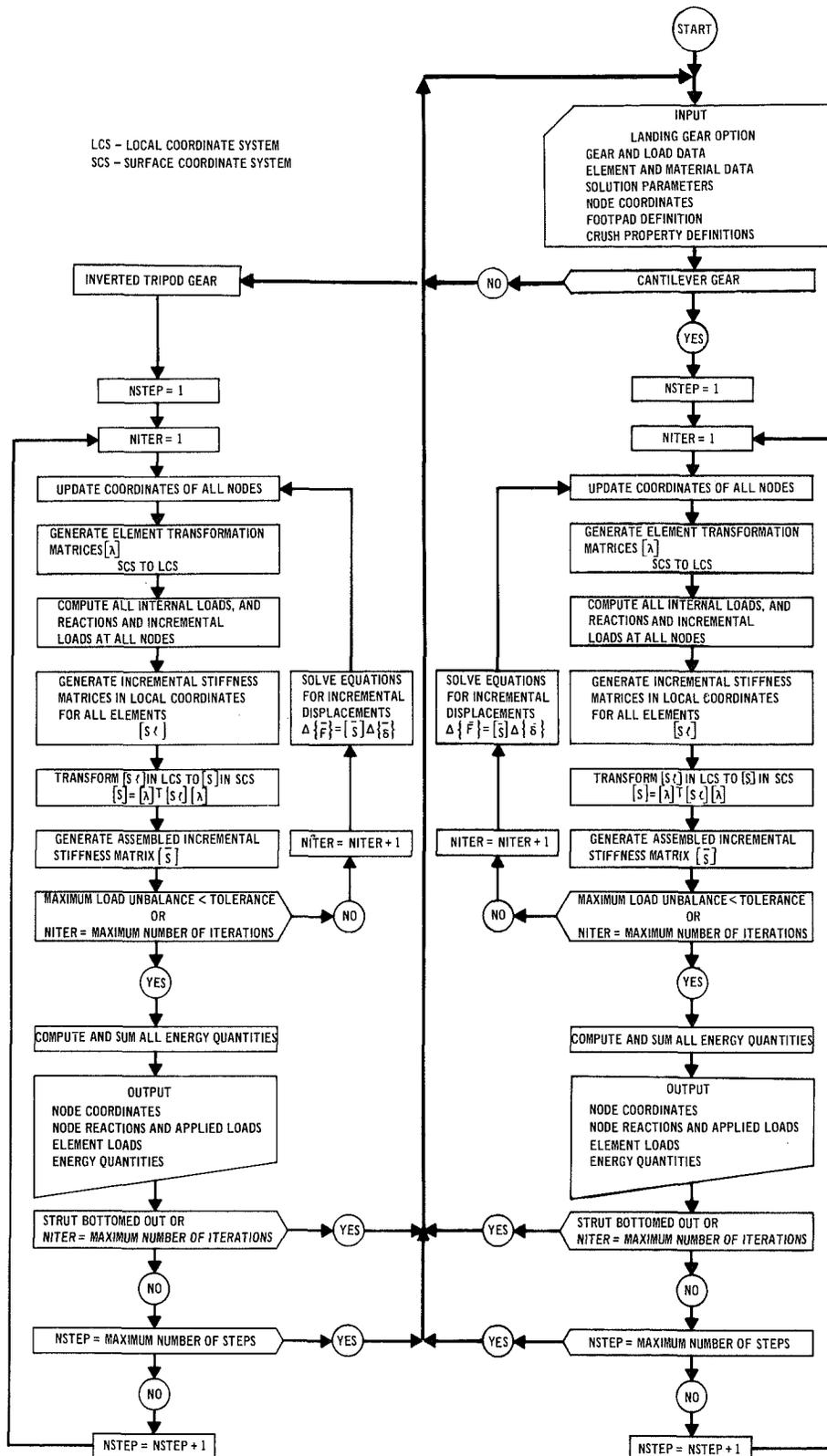


FIGURE 4-31 FLOW DIAGRAM STRUCTURAL ANALYSIS PROGRAM (Continued)

B.0 Comment Cards - As many comment cards as desired may be input by leaving columns 1 through 6 blank and entering the comments in columns 7 through 80. These comment cards may be located anywhere within the data cards as long as they precede the data terminator.

B.1 Joint Information Cards - Joint information is input on data cards in which a 1 is placed in column 1. Location of joint data in the fields of the card is indicated in Figure 4-32. Joints must be numbered sequentially (right-justified in columns 2 through 5) from 1 through the total number of joints (maximum of 74). The global coordinates, \bar{X} , \bar{Y} , and \bar{Z} , may be any right hand orthogonal coordinate system (see Figure 4-2). If the structure is planar, it will be simpler to choose a Global Coordinate System such that two of the axes lie in the plane of the structure. Advantage should be taken of any structural symmetry which may exist in selecting global axes.

Six data fields located between columns 36 and 53 are used to indicate various constraints at a particular joint. These constraints may be combinations of specified displacements (global deflections and/or rotations) and limits on reaction loads (global forces and/or moments) at a joint. Leaving these columns blank causes the program to assume that there are no displacement constraints at a joint; however, there are applied loads at the joint. These applied loads are discussed in the next paragraph. Constraints are defined by inserting nonzero integer identifying indicators (right-justified) in the fields of this region. The absolute value of these identifying indicators corresponds to the limit number on the code 3 data card defining the magnitudes of the particular constraint. When positive identifying indicators are used, the program assumes that upper and lower plastic limits are placed on the appropriate loads at a support and that the corresponding displacement or rotation is zero as defined on the related code 3 data card. For negative identifying indicators, the program assumes that no restraints are placed on the loads, but the displacements are specified on the related code 3 data card. The same code 3 data card may be used to define identical constraints at a number of joints. The maximum number of plastic loads and specified displacements is 88.

Applied forces are indicated by placing a separate set of integer identifying indicators in columns 54 through 57 (right-justified). The program looks for the three global components of force on cards with code number 4 in

HEADER CARD
(Header Card and Card Codes 0, 1, and 2)

STRUCTURE																																																																															
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80

(0) COMMENT CARDS

										COMMENTS																																																																					
BLANK																																																																															
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80

(1) JOINT INFORMATION CARDS

JOINT	JOINT COORDINATES										SUPPORT CONSTRAINTS				SPECIFIED LOAD VECTORS																																																																	
											DISP./FORCE		ROT./MOMENT		FORCE		MOMENT																																																															
	\bar{X}	\bar{Y}	\bar{Z}			\bar{X}	\bar{Y}	\bar{Z}			\bar{X}	\bar{Y}	\bar{Z}		F	N	O	M	E	N	T																																																											
1	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80

(2) REFERENCE POINT INFORMATION CARDS

REF POINT	REFERENCE POINT COORDINATES																																																																															
	\bar{X}	\bar{Y}	\bar{Z}																																																																													
2	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80

FIGURE 4-32 INPUT DATA FORMAT - CENTER BODY OPTION

column 1, and the corresponding identifying indicator in columns 2 through 5. Identifying force indicators must be sequential with a maximum of 74. Applied moments are numbered separately, with this set of identifying indicators being placed in columns 58 through 61, and the three corresponding global moment components being located on cards with a 5 in column 1. A total of 74 identifying moment indicators may be used. If either of these identifying indicators is zero (or blank), the appropriate applied load is zero and no corresponding code 4 or 5 data card is required. If displacements and/or rotations have already been restrained in specific directions, forces and/or moments may not be applied in these same directions.

Since free-free modes are obtained with the modal analysis routine, no constraints or applied loads should be specified when employing this program option. Thus, for modal analysis data cases, columns 36 through 61 should be left blank on all code 1 data cards.

B.2 Reference Point Information Cards - A code number 2 in column 1 indicates the data represents global coordinates and identifying joint number of a reference point. Reference points are provided to permit orientation of bar local coordinate (bending) axes, where structural joints will not suffice. Numbering of these reference points begins with one greater than the number of joints and is sequential through the total number of reference points. For example if there are 36 structural joints, the first reference point must be 37.

B.3 Force and Moment Limit Cards - A 3 in column 1 indicates data on the card is either a known displacement or rotation (columns 26 through 35), or upper and lower plastic force or moment constraints (columns 6 through 15 for an upper bound and 16 through 25 for a lower bound). The integer identifying number (limit number) in columns 2 through 5 corresponds to an identifying indicator or indicators on code 1 data cards, previously discussed. These limit numbers must be sequential in order. The sign of the identifying indicator on the code 1 data card determines what data are being read: A plastic force constraint and zero displacement (positive sign) or known displacement (negative sign). The known displacement may be nonzero or zero, simulating a conventional support. If the force is to be limited in the positive global direction, use the upper limit. A lower limit indicates a constraint on the

(Data Codes 3 Thru 6)

(3) FORCE/MOMENT LIMITS AND SPECIFIED DISPLACEMENT/ROTATION CARDS

LIMIT	LIMITS ON FORCE/MOMENT		SPECIFIED DISP/ROT VALUE
	UPPER LIMIT	LOWER LIMIT	
3	1-10	11-20	21-80

(4) SPECIFIED FORCE VECTOR CARDS

FORCE	SPECIFIED FORCE COMPONENTS	
	$F\bar{X}$	$F\bar{Y}$
4	1-10	11-20

(5) SPECIFIED MOMENT VECTOR CARDS

MOMENT	SPECIFIED MOMENT COMPONENTS	
	$M\bar{X}$	$M\bar{Z}$
5	1-10	11-20

(6) BAR INFORMATION CARDS

BAR NO.	BAR LOCAL AXIS LOCATORS		BAR CROSS-SECTIONAL AREA	BAR MOMENTS OF INERTIA		BAR MATERIAL MODULI		SHEAR FORM FACTORS		
	P	Q		R	A	I_N	I_T	J	E	G
6	1-10	11-20	21-30	31-40	41-50	51-60	61-70	71-80	81-90	91-100

FIGURE 4-33 INPUT DATA FORMAT - CENTER BODY OPTION

force in the negative global direction.

For modal analysis data cases, code 3 data cards should not be input.

B.4 Specified Force Vector Cards - Applied forces at a joint are input as three or less global force components on cards with a 4 in column 1. The identifying number (force number), right-justified in columns 2 through 5, corresponds to an identifying indicator on code 1 data cards thereby defining the joint or joints where forces will be applied.

For modal analysis data cases, code 4 data cards should not be input.

B.5 Specified Moment Vector Cards - Applied moments at a joint are input as three or less global moment components on cards with a 5 in column 1. The identifying number (moment number), right-justified in columns 2 through 5, corresponds to an identifying indicator on code 1 data cards to define the joint or joints where the moments will be applied.

Code 5 data cards should not be input for modal analysis data cases.

B.6 Bar Information Cards - A card with a 6 in column 1 indicates bar data are being input. Bar number is entered in columns 2 through 5, right-justified, with numbering being sequential from one through the total number of bars (130 maximum). Bar origin (point "p"), end (point "q"), and direction of the Y_ℓ bending axis (point "r"), locating the bar Local Coordinate System (Figure 4-1), are determined by specifying joint numbers in the appropriate locations in columns 6 through 14 (all right-justified). A reference point may be used for point "r," but points "p" and "q" must be nodes (joints) on the structure. Point "r" must not be on the line "pq" or its extension.

Bar cross-sectional area perpendicular to the "pq" (X_ℓ) axis, A; moment of inertia about the Z_ℓ axis, I_N ; moment of inertia about the Y_ℓ axis, I_T ; and torsional constant, J, are entered in the appropriate columns as indicated in Figure 4-33. Members pinned at both ends for bending about the Z_ℓ axis can be simulated by making the moment of inertia I_N equal to zero. Similarly making I_T equal to zero simulates a member pinned at both ends for bending about the Y_ℓ axis. If A is set equal to zero, no axial load will be carried by the member, and setting J equal to zero prevents the member from carrying torsion. Any combination of these may be utilized. Modulus of elasticity, E, and shear modulus, G, for the bar material are entered in columns 53 through 61 and 62 through 70, respectively. If E-format is used it must be right-justified.

Caution must be exercised when using pinned members or members with A , I_T , I_N , or J equal to zero, because a joint may be left with no load carrying capability in one of the three global directions or with no rotational restraint about one of the three global axes. An example of this would be two bars in the same plane pinned together. An artificial restraint (support) must be placed on the joint in the global direction in which there is no load carrying capability, even though there will be no force induced in this support direction. This instruction is input on code 1 data cards. The program will automatically reject any problem in which load carrying capability (translational or rotational) does not exist at any joint.

The program has the ability to account for the effect of bar shear strain on bending deflections. If a zero (blank) is placed in column 15, shear strain is not accounted for. A 1 in column 15 causes the program to read the values of shear form factors, K_N and K_T , and account for the effect of shear strain. The factor K_N is the total cross-sectional area divided by the area effective in carrying shear in the Y_ℓ direction, while K_T is the total area divided by the area effective in carrying shear in the Z_ℓ direction (see Figure 4-6).

The program can also idealize a bar pinned at one end and rigidly attached at the other. To exercise this option a zero must be placed in column 15. If the rotational release is to be for bending about the Y_ℓ axis (I_T), a 1 or 2 is placed in column 16. A 1 indicates the "p" end is pinned while a 2 indicates the "q" end is pinned. The same number code (1 - "p" end pinned, 2 - "q" end pinned) is used in column 17 if the rotational release is for bending about the Z_ℓ axis (I_N). Values inserted for K_N and K_T for these cases will be automatically utilized.

B.7 Rectangular Shear Web Information Cards - Shear web information is input on data cards in which a 7 is placed in column 1. Location of shear web data in the fields of the card is illustrated in Figure 4-34. Shear webs must be numbered sequentially from 1 through the total number of shear webs (maximum of 30). The integer representing the shear web number is right-justified in columns 2 through 5.

(Card Codes 7 and 8 and Data Option Indicator Card(s))

(7) RECTANGULAR SHEAR WEB INFORMATION CARDS

S H E A R W E B		S H E A R W E B J O I N T S										S H E A R W E B M A T E R I A L P R O P E R T I E S																																																																			
A N O		A	B	C	D	P					E					t																																																															
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80

(8) DATA TERMINATOR CARD

B L A N K																																																																															
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80

(\$INDATA) DATA OPTION INDICATOR CARD(S)

B L A N K																																																																															
																																																																											DATA OPTION INDICATORS (SEE FIGURE 4-35)				
																																																																											NAME 1 = XXX, NAME 2 = YYY,				
																																																																											\$END				
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80

FIGURE 4-34 INPUT DATA FORMAT - CENTER BODY OPTION

Four data fields located between columns 6 and 17 are used to specify the joints A, B, C, and D, numbered counterclockwise (see Figure 4-7), which represent the corners of the rectangular web. These integers must be right-justified in their respective fields. Care must be taken to insure that the joints A, B, C, and D define a plane and form a rectangle. The program will reject any problem in which this is not true.

Three data fields located between columns 18 and 47 are used to specify the material properties and thickness of the web. The Poisson's ratio, ν , modulus of elasticity, E, and thickness, t, of the web are input in these fields as indicated in Figure 4-34. These values must be right-justified in their fields if E-Format is used.

B.8 Data Terminator Card - After all data needed on code 1 through 7 data cards and the desired comment cards are input, a data terminator card is required to signify the end of the data. This card must contain an 8 in column 1 and may contain any desired comments in columns 7 through 80.

C. Data Option Indicator (NAMELIST) Card(s) - Following the previously described input data, a NAMELIST system of input indicated by \$INDATA is used to define option indicator cards for selection of output options, iteration method, modal analysis routine, program termination, and tape options. Figure 4-34 presents the card format to be used. \$INDATA is entered in columns 2 through 8 of the first data option card. Columns 10 through 68 on the first and any succeeding cards are used to specify optional data indicator values. \$END must be entered in columns 69 through 72 of the last data option card.

Data option indicators and their nominal and optional values are defined in Figure 4-35. If nominal values are acceptable for all NAMELIST items, \$INDATA is entered in columns 2 through 8, \$END is entered in columns 69 through 72, and the rest of the card left blank. By listing any of the option data indicators, such as INDNMA = 1, the nominal control is overridden, as in this case where normal mode analysis would be utilized. A data option indicator need be listed only if a value other than the nominal value is desired.

INDICATOR	NOMINAL VALUE	OPTIONAL VALUE	CONTROL FUNCTION
INDSFL	1	0	DO NOT _____ DO _____ } WRITE BAR LOCAL STIFFNESS AND TRANSFORMATION MATRICES.
INDSFG	1	0	DO NOT _____ DO _____ } WRITE BAR TRANSFORMED STIFFNESS MATRICES
INDISL	0	1	WILL NOT _____ WILL _____ } INPUT AN INITIAL SOLUTION OF DISPLACEMENTS AND ROTATIONS IN "SOLVEC",
INDITR	3X(NO. OF ROWS IN STRUCTURAL STIFFNESS MATRIX)	K (ANY INTEGER)	THE ITERATIVE SOLUTION OF THE SIMULTANEOUS EQUATIONS WILL BE STOPPED AFTER ("INDITR" + 1) ITERATIONS
ERRTOL	.0001	A (ANY REAL NUMBER)	THE ABSOLUTE VALUE OF "ERRTOL" IS USED AS THE ERROR TOLERANCE TO TERMINATE THE ITERATIVE SOLUTION FOR DISPLACEMENTS AND ROTATIONS.
SOLVEC	0. (i)	A (i)	THE INITIAL SOLUTION OF DISPLACEMENTS AND ROTATIONS IS ZERO. THE A i's (i = 1, 6X(NO. JOINTS)) ARE INPUT VALUES (SEPARATED BY COMMAS) DEFINING AN INITIAL SOLUTION OF DISPLACEMENTS AND ROTATIONS. (SOLVEC IS DIMENSIONED 444) ie. $A_{(1)} = \bar{X}$ DISPLACEMENT OF JOINT 1 . . . $A_{(4)} = \bar{X}$ ROTATION OF JOINT 1 . . . $A_{((j-1)X6 + 1)} = \bar{X}$ DISPLACEMENT OF JOINT j . . . $A_{((j-1)X6 + 4)} = \bar{X}$ ROTATION OF JOINT j.

FIGURE 4-35 NAMELIST (\$INDATA) DATA OPTION INDICATORS

INDICATOR	NOMINAL VALUE	OPTIONAL VALUE	CONTROL FUNCTION
RELAXF	1.	A ($1.0 \leq A \leq 2.0$)	EMPLOY GAUSS-SIEDEL SOLUTION METHOD THE RELAXATION FACTOR EMPLOYED IF THE OVERRELAXATION METHOD OF SOLUTION IS USED. RECOMMENDED OPTIONAL VALUE = 1.2
INDRLX	2	1	EMPLOY CONJUGATE GRADIENT SOLUTION TECHNIQUE. EMPLOY GAUSS-SIEDEL METHOD OF SOLUTION WITH NOMINAL RELAXATION FACTOR OF 1., OR USE OVERRELAXATION SOLUTION METHOD WITH OPTIONAL RELAXATION FACTOR DEFINED IN "RELAXF."
INDRKT	0	0	EMPLOY GAUSS-SIEDEL SOLUTION METHOD WITH AITKEN'S Δ^2 IMPROVEMENTS.
INDWKT	0	1	DO NOT ——— { READ STRUCTURAL STIFFNESS MATRIX, AND BAR LOCAL STIFFNESS AND TRANSFORMATION MATRICES FROM PHYSICAL TAPE 9. } DO ———
INDPLS	0	1	DO NOT ——— { WRITE STRUCTURAL STIFFNESS MATRIX AND BAR LOCAL STIFFNESS AND TRANSFORMATION MATRICES ON PHYSICAL TAPE 9. } DO ———
MINRST	6	K (ANY INTEGER)	DO NOT ——— { CONSIDER PLASTICITY THROUGH USE OF LIMITS ON FORCES/MOMENTS. } DO ——— LESS THAN "MINRST" RESTRAINTS WILL CAUSE PROGRAM TERMINATION DURING SOLUTION OF SIMULTANEOUS EQUATIONS.
INDNMA	0	1	DO NOT ——— { RUN NORMAL MODE ANALYSIS. } DO ———

FIGURE 4-35 NAMELIST (\$INDATA) DATA OPTION INDICATORS (Continued)

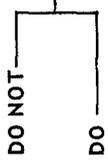
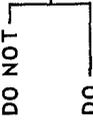
INDICATOR	NOMINAL VALUE	OPTIONAL VALUE	CONTROL FUNCTION
IRWKP	0 (i)	K (i) (INTEGERS)	ROW NUMBERS IN ASCENDING ORDER (SEPARATED BY COMMA'S) TO BE KEPT IN REDUCED STIFFNESS MATRIX WHEN RUNNING NORMAL MODE ANALYSIS (i = 1, 2, . . . , n WHERE n ≤ 102; "IRWKP" IS DIMENSIONED 103).
AMASS	0. (i)	A _i	DIAGONAL MASS MATRIX ELEMENTS (SEPARATED BY COMMAS) IN ASCENDING ORDER, INPUT WHEN RUNNING NORMAL MODE ANALYSIS (i = 1, 2, . . . , n WHERE n ≤ 102; "AMASS" IS DIMENSIONED 102).
IREDTO	ORDER OF STIFFNESS MATRIX	K (K ≤ 102)	THE REQUIRED ORDER OF REDUCED STIFFNESS MATRIX WHEN RUNNING NORMAL MODE ANALYSIS.
INDWNM	0	1	DO NOT  WRITE NORMAL MODE DATA ON TAPE FOR USE IN TASK ORDER TWO LANDING LOADS AND MOTIONS PROGRAM OR THE CENTER BODY LOADS PROGRAM. DO
TMAX	9999.	A (ANY REAL NUMBER)	NUMBER OF SECONDS AFTER WHICH MACHINE WILL PRINT PRESENT SOLUTION AND STOP, IF CONVERGENCE TO PRESCRIBED TOLERANCE HAS NOT BEEN REACHED (RECOMMENDED OPTIONAL VALUE = .9X(CP TIME)).
ISFDIM	12,904	K (INTEGER)	THE NUMBER OF WORDS OF STORAGE RESERVED TO STORE THE TOTAL STIFFNESS MATRIX. THE LENGTH OF LABELED COMMON, "CMAIN", MUST BE RE-DEFINED TO ALLOW STORAGE OF A STIFFNESS MATRIX WHICH REQUIRES "K" WORDS OF STORAGE SPACE, WHERE K > 12904.
INDWTS	0	1	DO NOT  PRINT THE NONZERO ELEMENTS OF THE TOTAL STIFFNESS MATRIX. DO
NEIGVL	5	M (5 ≤ M ≤ IREDTO-6)	NUMBER OF NON-RIGID BODY MODES TO BE CALCULATED IF MODAL ANALYSIS IS EMPLOYED, "NEIGVL" MUST BE A MULTIPLE OF 5.

FIGURE 4-35 NAMELIST (\$INDATA) DATA OPTION INDICATORS (Continued)

The ERRTOL control is important, since it affects computer time. The value specified, multiplied by 100, is the maximum acceptable percent error in the solution. The error for the Gauss-Siedel method, the Gauss-Siedel method with Aitken's Delta Squared improvements, and the overrelaxation method is defined as the largest relative difference between consecutive displacement and rotation solutions. This relative difference is calculated by taking the difference between given solution values for two successive iterations and dividing by the last value of the solution.

The error for the Conjugate Gradient Method is defined as the largest term found when the residual vector (see Reference (5)) is divided by the largest element in the force/moment vector. The program will run until it achieves the accuracy prescribed by ERRTOL or exceeds the maximum number of iterations (INDITR).

An initial set of nodal deflections may be input using the indicator SOLVEC. This option allows the program to start with these values for deflections and achieve convergence more rapidly. This option is also useful if minor changes are made in a structure after an initial solution has been obtained.

The maximum number of iterations allowed (INDITR), nominally three times the number of rows in the stiffness matrix, should be selected based on the size and sparsity of the stiffness matrix and desired length of computer run time. The nominal iteration method used is the Conjugate Gradient Method. If INDRLX is set equal to 0 the Gauss-Siedel method with Aitken's Delta Squared Improvements is employed. If INDRLX is set equal to 1, the Overrelaxation Method is employed and a value of RELAXF (the relaxation factor) between 1.0 and 2.0 should then be input. A nominal relaxation factor of 1.0 is set in the program, which is equivalent to a standard Gauss-Siedel solution (without Aitken's Delta Squared Improvements). For employing the Overrelaxation Method, a RELAXF equal to 1.2 has been found to result in minimum machine time and rapid convergence for a number of typical problems. Experience dictates that for highly redundant space frames, such as the legged lander,

the nominal solution technique (the Conjugate Gradient Method) is the best suited of the above methods and is recommended.

The MINRST option is selected based on the minimum number of restraints (supports) which must exist for stability (nominally six). If, at any time, less than this number of supports exist, the program will automatically terminate with an error message.

If it is desired to perform a normal mode analysis, INDNMA is set equal to 1. If reduction is to be employed, the required order of the reduced stiffness matrix is input in IREDTO. The row numbers to be retained in the reduced stiffness matrix must be listed in ascending order in IRWKP if reduction is employed. Values for the diagonal mass matrix terms associated with the degrees of freedom retained in the reduced stiffness matrix, are input in AMASS. If all six degrees of freedom at a particular joint are retained, the first three numbers in AMASS would represent the mass associated with this joint. The next three numbers would represent the mass moment of inertia associated with this joint relative to the directions of the global \bar{X} , \bar{Y} and \bar{Z} axes. If degrees of freedom at a joint are removed in the reduced stiffness matrix, the corresponding mass or inertia items should not be input in AMASS. If reduction is not required, no data is required for IRWKP and IREDTO and the number of terms in AMASS must equal the order of the system stiffness matrix.

The number of nonrigid body modes to be calculated is input in NEIGVL. NEIGVL, which has a nominal value of 5, can optionally be any multiple of five, and must be less than or equal to IREDTO less six.

Care must be exercised in the use of the reduction routine. It is possible to eliminate so many rows from the stiffness matrix that a number of the remaining degrees of freedom are no longer independent. This has the effect of eliminating one or more of the system's six rigid body modes. To insure that this does not occur for a general space frame, at least two translational degrees of freedom in each of three global coordinate directions should remain following reduction. For the special case of a straight beam, two torsional rotations should remain in addition to these translations.

Error messages are printed if the user accidentally specifies a reduction which leads to the elimination of too many rows from the stiffness matrix.

Should more than 12904 words be required to store the stiffness matrix in a given problem, ISFDIM must be set equal to the required number of words and labeled COMMON must be redimensioned accordingly.

4.3.1.2 Output Data - Center Body Option output includes all input data, NAMELIST indicator values used, nodal displacements and rotations, number of iterations required for convergence, maximum error, nodal forces and moments, bar forces and moments, shear flows in shear webs, and a CPU time summary. Modal analysis output includes all input data, NAMELIST indicator values used, natural frequencies, mode shapes, generalized inertia properties, and a CPU time summary. All input data is printed out in a block format. Classification of input data is by code number in column 1, as described in Section 4.3.1.1.

For problems not requiring modal analysis, nodal displacements and rotations are printed out. The number of iterations required for the solution as well as the maximum error in the solution are then printed. Global forces and moments acting on the joints are also output. Forces and moments acting on both ends of all bars are printed. The p, q, and r joint numbers used for each bar are also listed to aid in interpretation of the direction of these forces and moments (positive local sign convention is indicated in Figure 4-1). Shear flows and identifying joints are printed for all shear webs. Positive local sign convention for establishing directions of shear flows is given in Figure 4-7.

When plastic force constraints are violated, the program prints out the elastic solution, which constraints were violated, and the plastic solution (nodal deflections and rotations, and nodal forces and moments). Forces and moments acting on both ends of all bars as well as shear flows in all shear webs are printed out for the final solution. The last output item is a CPU time summary showing a breakdown of CPU time usage.

When employing the modal analysis routine, output data include: input data, the specified number of lowest natural frequencies (excluding rigid body modes), corresponding mode shapes (normalized) for both the reduced and complete systems, generalized inertia properties, and a CPU time summary.

On an optional basis, this modal analysis output may be placed on magnetic tape. This information is required by the Center Body Landing Loads Program, as discussed in Appendix E. In addition, the data on this tape is compatible with the Task Order Two Landing Loads and Motion Program, Reference (2). In this case, the Task Order Five modal analysis could be employed for a footpad structure of a platform lander. Thus the improved matrix reduction routine could be taken advantage of.

4.3.1.3 Example of Program Operation - Examples of typical structural and modal analysis problems are included to illustrate interpreting input instructions and output data.

An example of obtaining deflections and internal loads for a center body structure subject to external loads is presented in Section 6.2.4 for the Task Order Three lander center body subject to typical landing loads. Input data and output listing for this problem are presented in Appendix A.

Input data and resulting output data when employing the modal analysis routine of the Center Body Option is given in Appendix B. This analysis was conducted on the center body structure of the Task Order Three lander. The idealization of the center body structure and plots of a number of the mode shapes are presented in Section 6.2.1. This modal data was input to the Landing Loads and Motions Program when studying the effects of center body flexibility on correlation with drop test data for this lander. This landing analysis is discussed in Section 6.2.2.

The data set shown in Appendix B is for a structural idealization with 33 joints and 53 bar members. This results in a stiffness matrix of order 198. The matrix reduction routine was employed to reduce this to order 99. This was accomplished by removing the rotations at all of the joints. Thus, the input mass matrix contained terms associated with each of the three translations at each joint. Twenty modes were requested from the modal analysis routine. Only the mode shapes for the complete system are shown in Appendix B.

4.3.2 Landing Gear Option - This section includes definition of input requirements and format, and output interpretation for the Landing Gear Option of the Structural Analysis Program. Examples of input and output data for a typical problem are contained in Appendix C.

4.3.2.1 Input Data - For each Landing Gear Option data set a data case

header card and information describing landing gear type, loading information, strut properties, gear geometry, footpad information, energy absorption requirements, and indicators needed to control program operation are required as input data. Multiple data cases may be run by stacking data sets. Each data set must consist of a header card followed by the appropriate data cards.

A. Header Card - A header card is required as the first card of a data set to indicate that the Landing Gear Option will be employed. This card must contain the characters GEAR in columns 1 through 4 as indicated in Figure 4-36.

B. Data Cards - The input format for data cards employs a system of code numbers, right-justified in columns 1 and 2, to identify the type of data being read. The seventeen code numbers and corresponding type of input information are as follows:

- (0) Comment cards
- (1) Gear and load card
- (2) Friction card
- (3) Applied displacement card
- (4) Strut material cards
- (5) Solution parameter card
- (6) Nodal point cards
- (7) Footpad card
- (8) Material parameter cards
- (9) Material crush cards
- (10) Compression crush distance cards
- (11) Tension crush distance cards
- (12) Compression spring rate cards
- (13) Tension spring rate cards
- (14) Compression plastic force cards
- (15) Tension plastic force cards
- (16) Data terminator card

B.0 Comment Cards - As many comment cards as desired may be input by leaving columns 1 through 6 blank and entering comments in columns 7 through 80 as shown in Figure 4-36. These comment cards may be located anywhere within the data cards and must precede the data terminator card.

(Header Card and Card Codes 0, 1, and 2)

HEADER CARD																																																																																		
GEAR		1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	
(0) COMMENT CARDS																																																																																		
BLANK																																																																																		
(1) GEAR AND LOAD CARD																																																																																		
BLANK	BLANK	BLANK	L	LOAD	INDICATOR	BLANK	LENGTH	OF	STRU	T	2	EULER ANGLES																																																																						
												ψ	θ	ϕ	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68
(2) FRICTION CARD																																																																																		
BLANK	BLANK	BLANK	APPLIED	NORMAL	DISPLACEMENT	FRICTION	COEFFICIENT																																																																											
								1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75

FIGURE 4-36 INPUT DATA FORMAT - LANDING GEAR OPTION

B.1 Gear and Load Card - Information describing landing gear type, type of applied loading, strut parameter for the cantilever gear, and the orientation of the landing surface is input on a data card in which a 1 is placed in column 2. Location of data fields on this card is illustrated in Figure 4-36. A 1 right-justified in columns 6 through 10 signifies that the gear being analyzed is an inverted tripod. Similarly, a 2 in this field indicates a cantilever gear.

A 1 right-justified in columns 11 through 15 indicates that the applied loading is of the frictional type in which the displacement normal to a landing surface is specified. For this type of loading, a code 2 data card is required as input data to specify the applied normal displacement and the friction coefficient. A 2 right-justified in columns 11 through 15 indicates that three applied displacement components are specified. For this case an associated code 3 data card must be input. As explained in Section B.2, a code 2 data card must also be input to specify the direction normal to the landing surface if the footpad has attenuation.

For the cantilever gear only, the length of strut 2 (see Figure 4-17) must be input in columns 21 through 30. This information is required to locate node 4 on the cantilever gear. For the inverted tripod gear this data field is left blank.

Three data fields located between columns 31 and 60 are used to specify the Euler angles ψ , θ , and ϕ which orient the Surface Coordinate System relative to the Lander Coordinate System. These Euler angles must be input in degrees. A definition of these angles and the associated sign convention is given in Figure 4-20.

B.2 Friction Card - Information describing the direction and magnitude of applied displacement (normal to the landing surface) at the footpad joint and the coefficient of friction of the surface is input on a data card in which a 2 is placed in column 2. Location of data fields on this card is illustrated in Figure 4-36. A 1, 2, or 3 right-justified in columns 6 through 10 of this card signifies that the positive direction normal to the landing surface is the X, Y, or Z surface coordinate direction, respectively. This information as well as the information supplied in the remaining data fields of this card must be specified if loading of the friction type is selected

on the code 1 data card. Should the three applied displacement components case be specified on the code 1 data card and the footpad have at least 1 level of attenuation, the normal displacement direction indicator must be supplied on the code 2 data card to indicate the surface coordinate direction to check for footpad crushing. For this case, the remaining information on the data card is not used. Care must be exercised when selecting the coordinate system to insure that the normal surface coordinate direction is out of the surface.

The magnitude of the applied normal displacement in the surface coordinate direction indicated in columns 6 through 10 is supplied in columns 21 through 30. The coefficient of friction of the landing surface is entered in columns 31 through 40.

B.3 Applied Displacement Card - For the case in which three displacement components at the footpad joint are specified, the three surface coordinate components of applied displacement are entered on a data card with a 3 in column 2. Location of data fields on this card is illustrated in Figure 4-37. Three data fields located between columns 21 and 50 are used to enter the three components of applied displacement at the footpad joint. These displacement components must be given in the Surface Coordinate System.

B.4 Strut Material Cards - Each strut of a gear must be assigned a material identification number to indicate which material, input on code 8 through 15 data cards, makes up the strut. The material identification number of each element (strut) is input on data cards with a 4 in column 2. Location of data fields on this card is illustrated in Figure 4-37. One card must be input for each element of the gear, i.e., three for the inverted tripod and four for the cantilever.

The element number must be right-justified in columns 6 through 10. The number identifying the material of which this element is made is right-justified in columns 11 through 15. More than one element may have the same material identification number. This is convenient in the case of a gear which has identical drag struts since then only one set of material properties has to be defined for these struts.

(Card Codes 3 Thru 6)

(3) APPLIED DISPLACEMENT CARD

BLANK										APPLIED DISPLACEMENT COMPONENTS IN SURFACE COORDINATES																																																																					
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80
BLANK										X										Y										Z																																																	

(4) STRUT MATERIAL CARDS

BLANK										ELEMENT										MATERIAL																																																											
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80

(5) SOLUTION PARAMETER CARD

BLANK										NO. OF LOADING STEPS										NO. OF DIFFERENT MATERIALS										ENERGY CUTOFF										SOLUTION TOLERANCE																																							
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80
BLANK										I N D E N T I F I C A T O R										B L A N K										SOLUTION TOLERANCE																																																	

(6) NODAL POINT CARDS

BLANK										NODE NO.										NODE COORDINATES																																																											
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80
BLANK										X										Y										Z																																																	

FIGURE 4-37 INPUT DATA FORMAT - LANDING GEAR OPTION

B.5 Solution Parameter Card - The number of steps for investigating the applied loading, the number of iterations to be allowed, the number of different materials used in the gear idealization, an indicator to specify in which direction to check the energy absorbed, the energy absorption cutoff, and the solution tolerance are input on a data card with a 5 in column 2. Location of data fields on this card is illustrated in Figure 4-37.

Four data fields located between columns 6 and 25 are used to specify the number of steps to divide the applied loading, the number of iterations allowed before the solution is assumed to be nonconvergent, the number of different materials employed in idealizing the struts of the gear, and an indicator to specify in which surface coordinate direction to check the energy absorption associated with the loads acting on the footpad. These integers must be right-justified in their respective fields. The energy check indicator is set to 1, 2, or 3 if the energy absorption is to be checked in the X, Y, or Z surface coordinate direction, respectively.

The energy cutoff used to make the energy absorption check is input in columns 31 through 40. In assigning a sign to this cutoff it must be remembered that a plus sign signifies that the force and corresponding displacement components of interest are in the same direction while a minus sign signifies that they are opposite in direction. The tolerance used to check for solution convergence is input in columns 41 through 50. For input data employing units of centimeters and dynes, the solution would be assumed converged to a tolerance specified as 4.0×10^4 when the magnitude of the largest unbalance, between any internal and external load component at any joint of the gear, was less than 4.0×10^4 dynes or 4.0×10^4 dyne-cm as the case may be. For each loading step, the program will run until the solution converges to the tolerance of the maximum number of iterations is exceeded.

B.6 Nodal Point Cards - For each node point (joint) of the gear, except for node 4 of the cantilever gear, the initial node coordinates in the Lander Coordinate System are input on cards with a 6 in column 2. The location of data fields on these cards is illustrated in Figure 4-37. Four of these cards must be input for both the inverted tripod and cantilever gears. On each card, the node point number must be right-justified in columns 6 through 10. Three data fields located between columns 21 and 50 are used to input the X, Y, and Z coordinates of the node in the Lander Coordinate System.

B.7 Footpad Card - Information regarding attenuation on the bottom of the footpad is input on a card with a 7 in column 2. Location of data fields on this card is illustrated in Figure 4-38. An attenuation indicator to signify whether or not the footpad has attenuation is right-justified in columns 6 through 10. A 1 signifies that the footpad has attenuation while a 2 indicates that there is none. The number of attenuation crush levels is entered, right-justified, in columns 11 through 15.

Three data fields located between columns 21 and 50 are used to input the magnitudes of up to three attenuation crush forces (see Figure 4-28). These crush forces must be entered in the order of increasing magnitude. The crush stroke magnitudes, associated with up to three crush levels, are entered in three data fields located between columns 51 and 80. Each stroke entry represents the stroke of the footpad honeycomb during crushing of the level of interest and does not include the stroking that may have occurred due to crushing of lower levels. For a footpad without attenuation the information in columns 21 through 80 will be ignored.

B.8 Material Parameter Cards - For each different material, the material identification number, plasticity indicator for axial displacements, moduli of elasticity, and cross-sectional properties are input on a card with an 8 in column 2. Location of data fields on this card is illustrated in Figure 4-38. Material identification number is entered, right-justified, in columns 6 through 10. A plasticity indicator for this material is entered, right-justified, in columns 11 through 15. A plasticity indicator of 1 signifies that for axial displacements the material behaves elastically. A plasticity indicator of 2 indicates that for axial displacements the material behaves plastically. That is, the axial characteristics of this material are those of honeycomb attenuation.

Two data fields located between columns 21 and 40 are used to input the moduli of elasticity for bending and axial displacements of the strut composed of the material identified in columns 6 through 10. Two data fields located between columns 41 and 60 are used to input the moment of inertia for bending displacements and the cross-sectional area of the strut composed of the material identified in columns 6 through 10.

For a material with plastic axial capability (a plasticity indicator of 2), the modulus of elasticity for bending displacements and the moment of

(Card Codes 7 Thru 10)

(7) FOOTPAD CARD

INDICATOR										BLANK										NO. CRUSH LEVELS										BLANK										ATTENUATION CRUSH FORCES										ATTENUATION CRUSH STROKES																													
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80

(8) MATERIAL PARAMETER CARDS

P L A T I N O										I N D I C A T O R										B L A N K										MODULI OF ELASTICITY										CROSS SECTIONAL PROPERTIES																																							
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80

(9) MATERIAL CRUSH CARDS

P L A T I N O										I N D I C A T O R										B L A N K										ALLOWABLE STROKE										UNLOADING SPRING RATE																																							
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80

(10) COMPRESSION CRUSH DISTANCE CARDS

P L A T I N O										B L A N K										CRUSH DISTANCES IN COMPRESSION																																																											
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80

FIGURE 4-38 INPUT DATA FORMAT - LANDING GEAR OPTION

inertia must be entered if the strut composed of this material can carry bending loads. These parameters should not be entered if the strut cannot carry bending loads. For either of the above materials, the modulus of elasticity for axial displacements and the cross-sectional area will be ignored since this information regarding axial capability is input in the crush property cards (card codes 9 through 15) for a material with plastic axial capability. The material for the lower member of the main strut (element 4 in Figure 4-17(b)) of the cantilever gear is an example of a material with plastic axial capability for which the bending properties must be input. The material for the main strut (element 2 in Figure 4-17(a)) of the inverted tripod gear is an example of a material with plastic axial capability for which bending properties would not be input.

For a material with elastic axial capability (a plasticity indicator of 1), the modulus of elasticity for axial displacements and the cross-sectional area must be entered. The modulus of elasticity for bending displacements and the moment of inertia are input only if the associated strut has bending capability. The material for the upper member of the main strut (element 2 in Figure 4-17(b)) of the cantilever gear is an example of a material with elastic axial capability for which bending properties must be input. For an inverted tripod gear with rigid drag struts, the material for the drag struts (elements 1 and 3 in Figure 4-17(a)) is an example of a material with elastic axial capability for which bending properties would not be input.

B.9 Material Crush Cards - For each material with plastic axial capability (a material which has a plasticity indicator of 2 on the code 8 data card), the material identification number, crush indicators, allowable strokes, and unloading spring rates must be input on a data card with a 9 in column 2. The location of data fields on this card is illustrated in Figure 4-38. The material identification number is right-justified in columns 6 through 10. An unloading indicator is right-justified in columns 11 through 15. A value of 0 for this indicator signifies that unloading of this material will be along a slope specified by the spring rates input in columns 41 through 60 of this card. An unloading indicator of 1 signifies that the material will unload along a slope specified by the next spring rate associated with the load-stroke curve. An initial stroke indicator is input, right-justified, in columns 16 through 20.

A value of 1 for this indicator signifies that the strut which is composed of this material will initially stroke in tension. A value of -1 indicates that this strut will initially stroke in compression. This information is needed for a crushable material to indicate whether the stiffness of the strut at the start of the loading should reflect the first spring rate in tension or that in compression.

Two data fields located between columns 21 and 40 are used to input the magnitudes of allowable (bottoming) strokes in compression and tension. For the material of a strut which can stroke in one direction only, the appropriate bottoming stroke should be input as 1.E-20. Two data fields located between columns 41 and 60 are used to input the unloading spring rate magnitudes in compression and tension. These values should be input only if the unloading indicator is input as 0 in columns 11 through 15.

B.10 Compression Crush Distance Cards - For each material with plastic axial capability, the crush distances in compression (see Figure 4-18) must be input on a data card with a 10 in columns 1 and 2. Data fields on this card are defined in Figure 4-38. The material identification number is right-justified in columns 6 through 10.

Five data fields, located between columns 21 and 70, are used to define the magnitudes of up to five crush distances in compression. Any number of crush distances between one and five may be input. The last crush distance input must be greater than the allowable compression stroke input on a code 9 data card for this material. This crush distance should have a fictitious value of 1.E+20 to insure that it will not be exceeded during program iteration prior to final convergence. For the material of a strut which can stroke only in tension, one compressive crush distance should be input at a value of 1.E+20.

B.11 Tension Crush Distance Cards - For each material with plastic axial capability, the crush distances in tension (see Figure 4-18) must be input on a data card with an 11 in columns 1 and 2. Data fields on this card are defined in Figure 4-39. The material identification number is right-justified in columns 6 through 10.

Five data fields, located between columns 21 and 70, are used to define the magnitudes of up to five crush distances in tension. Any number of crush distances between one and five may be input. The last crush distance input

(Card Codes 11 thru 14)

(11) TENSION CRUSH DISTANCE CARDS

P L A S T I C											B L A N K											C R U S H D I S T A N C E S I N T E N S I O N																																																									
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80
11																						5TH																																																									

(12) COMPRESSION SPRING RATE CARDS

P L A S T I C											B L A N K											S P R I N G R A T E S I N C O M P R E S S I O N																																																									
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80
12																						5TH																																																									

(13) TENSION SPRING RATE CARDS

P L A S T I C											B L A N K											S P R I N G R A T E S I N T E N S I O N																																																									
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80
13																						5TH																																																									

(14) COMPRESSION PLASTIC FORCE CARDS

P L A S T I C											B L A N K											P L A S T I C F O R C E S I N C O M P R E S S I O N																																																									
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80
14																						5TH																																																									

FIGURE 4-39 INPUT DATA FORMAT - LANDING GEAR OPTION

must be greater than the allowable tension stroke input on a code 9 data card for this material. This crush distance should have a fictitious value of 1.E20 to insure that it will not be exceeded during program iteration prior to final convergence. For the material of a strut which can stroke only in compression, one tension crush distance should be input at a value of 1.E+20.

B.12 Compression Spring Rate Cards - For each material with plastic axial capability, the spring rates in compression (see Figure 4-18) must be input on a card with a 12 in columns 1 and 2. Data fields on this card are defined in Figure 4-39. The material identification number is right-justified in columns 6 through 10.

Five data fields located between columns 21 and 70 are used to define the magnitudes of up to five spring rates in compression. Any number of spring rates between 1 and 5 may be input. For the material of a strut which can only stroke in tension, one spring rate in compression should be input at a value equal to the bottoming spring rate.

B.13 Tension Spring Rate Cards - For each material with plastic axial capability, the spring rates in tension (see Figure 4-18) must be input on a data card with a 13 in columns 1 and 2. Data fields on this card are defined in Figure 4-39. The material identification number is right-justified in columns 6 through 10.

Five data fields located between columns 21 and 70 are used to define the magnitude of up to five spring rates in tension. Any number of spring rates between 1 and 5 may be input. For the material of a strut which can only stroke in compression, one spring rate in tension should be input at a value equal to the bottoming spring rate.

B.14 Compression Plastic Force Cards - For each material with plastic axial capability, the plastic forces in compression (see Figure 4-18) must be input on a data card with a 14 in columns 1 and 2. Data fields on this card are defined in Figure 4-39. Material identification number is entered, right-justified, in columns 6 through 10.

Five data fields located between columns 21 and 70 are used to define the magnitudes of up to five plastic forces in compression. Any number of plastic forces between 1 and 5 may be input. For the material of a strut which can only stroke in tension, one plastic force in compression should be input at

a value slightly less than the product of SRC(1) and CDC(1).

B.15 Tension Plastic Force Cards - For each material with plastic axial capability, the plastic forces in tension (see Figure 4-18) must be input on a data card with a 15 in columns 1 and 2. Data fields on this card are defined in Figure 4-40. Material identification number is right-justified in columns 6 through 10.

Five data fields located between columns 21 and 70 are used to define the magnitudes of up to five plastic forces in tension. Any number of plastic forces between 1 and 5 may be input. For the material of a strut which can only stroke in compression, one plastic force in tension should be input at a value slightly less than the product of SRT(1) and CDT(1).

B.16 Data Terminator Card - After all data needed on code 1 through 15 data cards and the desired comment cards are input, a data terminator card is required to signify the end of the data set. This card must contain a 16 in columns 1 and 2.

4.3.2.2 Output Data - Output data from the Landing Gear Option includes all input data, initial conditions, displaced conditions for the gear at each step of the specified loading, and a CPU time summary.

The initial conditions printed for the gear include the X, Y, and Z coordinates of all nodes in the Surface Coordinate System.

The displaced conditions, printed for the gear at each step of loading, include the following: step of loading; number of iterations required to reach convergence; coordinates of all nodes in the Surface Coordinate System; external forces and moments at the nodes in the Surface Coordinate System; the internal axial force, stroke, and axial stiffness of each element; the internal shear force and moment at each end of all elements; the total energy absorbed at the footpad in both surface and lander coordinate components; the number of remaining (noncrushed) footpad attenuation levels; and surface and lander coordinate components of the energy absorbed by footpad attenuation during the loading step concerned.

Should a particular loading step fail to converge, the solution existing in the program, after the maximum number of iterations had been employed, would be printed out and program termination would occur. Should the allowable stroke in any of the members be exceeded or the load normal to the footpad be

(Card Codes 15 and 16)

(15) TENSION PLASTIC FORCE CARDS

P L A S T I C										B L A N K										P L A S T I C																																																											
B L A N K										B L A N K										P L A S T I C																																																											
15										15										15																																																											
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80

(16) DATA TERMINATOR CARD

16										16										16																																																											
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80

FIGURE 4-40 INPUT DATA FORMAT - LANDING GEAR OPTION

tensile for any loading step, the current solution and an appropriate error message would be printed, and program termination would occur. The program will continue through all loading steps unless one of the above errors occurs causing termination. If at any loading step the energy cutoff in the appropriate surface coordinate direction is exceeded, a message will be printed to indicate this fact and the program will continue.

4.3.2.3 Example of Program Operation - To illustrate interpretation of input instructions and output data, a typical example of the Landing Gear Option is included. An example problem where the inverted tripod gear of the Task Order Three lander was analyzed is discussed in Section 6.1.1. Input data and output listing for this example are presented in Appendix C.

5. LANDING LOADS AND MOTIONS PROGRAM

The Landing Loads and Motions Program provides the capability of predicting the landing dynamics of a legged lander. Program output presents the time histories of the landing gear loads and the spatial positions, velocities, and accelerations of the lander. Options are available for determining the distribution of landing loads throughout the lander structure and lander stability information.

The lander is comprised of up to five landing gears connecting the center body to the footpads which make contact with the landing surface. The center body may be treated as either a rigid structure with up to six rigid body degrees of freedom, or the effects of a flexible structure may be superimposed on these rigid body motions. Inverted tripod or cantilever landing gears having elastic-plastic load-stroke characteristics, velocity dependent energy absorption characteristics, or a combination of the two may be considered. Two soil mechanics routines are available and, on an optional basis, footpad attenuation material may be located on the bottom of each footpad.

Analytical methods developed for this program are presented in Section 5.1. Organization of the computer program is presented in Section 5.2 and operating instructions are discussed in Section 5.3. A program listing is given in Appendix I.

5.1 Analytical Methods - Analytical methods associated with the Landing Loads and Motions Program are presented below. Included are discussions of coordinate systems, equations of motion, structural idealization, soil mechanics routines, and footpad attenuation system.

5.1.1 Coordinate Systems - Four types of coordinate systems, as shown in Figure 5-1, are used to locate the lander as a function of time. These consist of two axis systems fixed relative to the planet, and two systems moving with the lander. These coordinate systems are all right-handed, and each has three orthogonal axes.

The coordinate systems are defined as follows:

Surface Coordinate System (X_s, Y_s, Z_s) - A coordinate system fixed in the planet and oriented with respect to the slope of the local

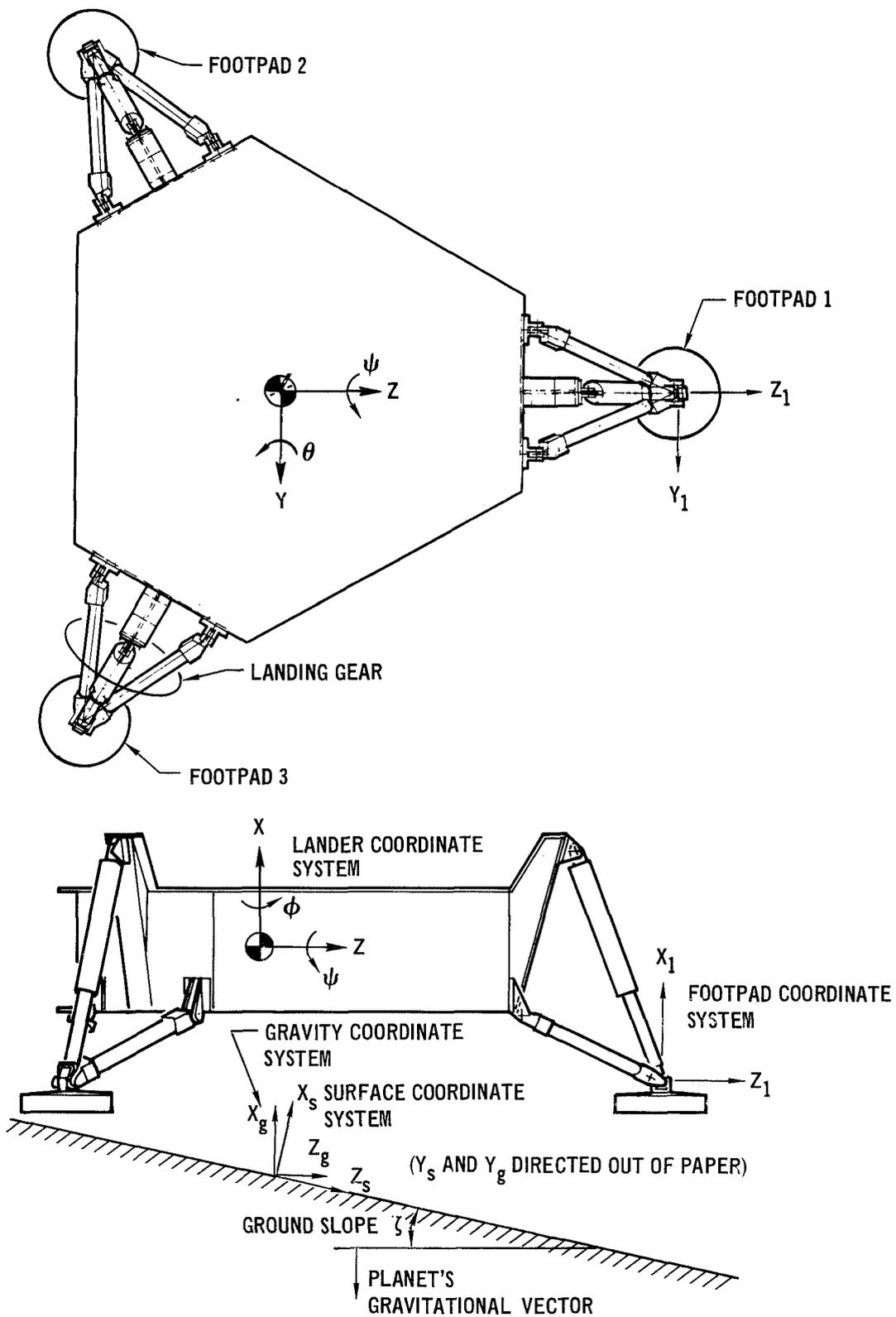


FIGURE 5-1 COORDINATE SYSTEMS

surface. The X axis of this system is perpendicular to the ground surface. The Z axis corresponds to the principle direction along the ground slope and the Y axis is 90 degrees across the slope.

Gravity Coordinate System (X_g, Y_g, Z_g) - A coordinate system fixed relative to the planet's local acceleration of gravity vector. The X_g axis is directed opposite to the local gravitational vector and the Y_g axis is parallel to the Surface Coordinate System Y_s axis. The Z_g axis is perpendicular to the gravitational vector and in the same general direction as the Surface Coordinate System Z_s axis.

Lander Coordinate System (X, Y, Z) - A coordinate system moving with the lander and fixed at the center of gravity of the center body. The lander's angular positions are defined in terms of the three Euler angles ψ , θ , and ϕ . Definition of the lander's angular position relative to the landing surface is based on a specific order for these Euler angle rotations. The order required for these rotations consists of an initial rotation, ψ , about the Z_s axis, followed by a rotation, θ , about the displaced Y axis and finally by a rotation, ϕ , about the direction of the displaced X axis resulting from the previous two rotations. Note, that when all three of these Euler angles are zero, the Lander Coordinate System is aligned with the Surface Coordinate System.

Footpad Coordinate System (X_i, Y_i, Z_i) - A coordinate system moving with the ith footpad and fixed at the footpad-main strut pivot point. The $Y_i - Z_i$ plane remains parallel to the bottom of the footpad as the footpad impacts the landing surface. Footpad 1 may be located anywhere relative to the Lander Coordinate System. The remaining footpads are numbered consecutively in a positive angular direction about the X axis.

The Lander Coordinate System is related to the Surface Coordinate System by the following expression:

$$\begin{pmatrix} X_s \\ Y_s \\ Z_s \end{pmatrix} = \begin{bmatrix} DC11 & DC12 & DC13 \\ DC21 & DC22 & DC23 \\ DC31 & DC32 & DC33 \end{bmatrix} \begin{pmatrix} X \\ Y \\ Z \end{pmatrix} \quad (5-1)$$

The elements in the lander's direction cosine matrix, [DC] are given as follows:

$$\begin{aligned}
 DC11 &= \text{Cos } \theta \text{ Cos } \psi \\
 DC12 &= \text{Sin } \phi \text{ Sin } \theta \text{ Cos } \psi - \text{Cos } \phi \text{ Sin } \psi \\
 DC13 &= \text{Cos } \phi \text{ Sin } \theta \text{ Cos } \psi + \text{Sin } \phi \text{ Sin } \psi \\
 DC21 &= \text{Cos } \theta \text{ Sin } \psi \\
 DC22 &= \text{Sin } \phi \text{ Sin } \theta \text{ Sin } \psi + \text{Cos } \phi \text{ Cos } \psi \\
 DC23 &= \text{Cos } \phi \text{ Sin } \theta \text{ Sin } \psi - \text{Sin } \phi \text{ Cos } \psi \\
 DC31 &= -\text{Sin } \theta \\
 DC32 &= \text{Sin } \phi \text{ Cos } \theta \\
 DC33 &= \text{Cos } \phi \text{ Cos } \theta
 \end{aligned} \tag{5-2}$$

In addition to the above transformations, relationships between the time derivatives of the Euler angles and the angular velocity components about the lander coordinate axes are required. Integration of these define the lander's angular position as a function of time. The required relationships are expressed as

$$\begin{aligned}
 \dot{\phi} &= \dot{\Omega}_x + \text{Tan } \theta (\dot{\Omega}_y \text{ Sin } \phi + \dot{\Omega}_z \text{ Cos } \phi) \\
 \dot{\theta} &= \dot{\Omega}_y \text{ Cos } \phi - \dot{\Omega}_z \text{ Sin } \phi \\
 \dot{\psi} &= (\dot{\Omega}_z \text{ Cos } \phi + \dot{\Omega}_y \text{ Sin } \phi) / \text{Cos } \theta
 \end{aligned} \tag{5-3}$$

where $\dot{\Omega}_x$, $\dot{\Omega}_y$, and $\dot{\Omega}_z$ are the components of the lander's angular velocity in the Lander Coordinate System.

5.1.2 Equations of Motion - The lander's equations of motion are discussed in two parts. The first presents the rigid body equations for the footpads considered to be in contact with the landing surface. The second part presents the development of the center body equations of motion, including the effects of a flexible center body structure.

5.1.2.1 Contacting Footpads - A footpad is termed "contacting" for two conditions. In the first, a footpad is considered to be contacting only after the footpad actually makes contact with the landing surface. Footpads not considered in contact with the landing surface are included in the center body equations as discussed in Section 5.1.2.2.

Secondly, on an optional basis, all of the lander's footpads may be treated as contacting footpads even if some of the footpads have not yet impacted the landing surface. In this case, inertia loading of the landing gear struts, due to the footpad mass, is simulated.

Each footpad considered to be in contact with the landing surface has three rigid body translation degrees of freedom. Two of these are in the plane of the landing surface and the third is normal to the landing surface. Equations of motion for each contacting footpad are written in the form

$$\begin{bmatrix} m_i & 0 & 0 \\ 0 & m_i & 0 \\ 0 & 0 & m_i \end{bmatrix} \begin{Bmatrix} \ddot{X}_i \\ \ddot{Y}_i \\ \ddot{Z}_i \end{Bmatrix} = \begin{Bmatrix} F_{x_i} \\ F_{y_i} \\ F_{z_i} \end{Bmatrix} + \begin{Bmatrix} -m_i g \cos \zeta \\ 0 \\ m_i g \sin \zeta \end{Bmatrix} \quad (5-4)$$

Terms in Equation (5-4) are defined as follows:

- m_i = mass of ith footpad.
- $\ddot{X}_i, \ddot{Y}_i, \ddot{Z}_i$ = translational accelerations of ith footpad in Surface Coordinate System.
- $F_{x_i}, F_{y_i}, F_{z_i}$ = sum of forces acting on ith footpad in Surface Coordinate System.
- g = local acceleration of gravity.
- ζ = local surface slope.

Two sets of forces are acting on each contacting footpad. These are the loads due to the stroking of the landing gear struts and the loads resulting from the interaction between the soil and the footpad attenuation system. These soil loads are zero when the footpad is clear of the landing surface.

5.1.2.2 Center Body - The effects of a flexible center body structure on the center body's motion have been included in the analysis. This is an optional feature of the Landing Loads and Motions Program and may be suppressed through input data. If suppressed, the center body is treated as a rigid body with up to six degrees of freedom.

To completely describe the dynamic motion of the elastic center body, a continuous elastic body must be considered. However, for the analysis of complex structures, the structure is often idealized as a network of finite elements. Motions of the idealized structure are determined at a finite number

of arbitrarily selected control points distributed throughout the body. The motion of each control point on this body is expressed in terms of three translational and three rotational displacements. This idealization results in a total number of equations of motion equal to six times the number of control points selected.

In order to reduce the number of equations to be solved in the Landing Loads and Motions Program, the normal mode method, References (9) and (10), was employed. In this method the motion of the center body is approximated by the combination of a limited number of vibratory modes plus the six rigid body modes. By selecting the vibratory modes which will be excited for a given landing condition, the behavior of the flexible center body structure is obtained.

For the analysis of the legged lander, the center body's free-free (unrestrained) modes were chosen as the vibratory modes. The rigid body modes were assumed to be the three translational displacements defining the position of the center body center of gravity in the Surface Coordinate System and three angular displacements defined in the Lander Coordinate System.

In developing the center body equations of motion, expressions defining the motion of a point on the center body were obtained. These were used to evaluate the kinetic and potential energy of the center body. The final form of the center body's equations of motion were obtained by applying the Lagrangian equations to these energy expressions.

The total displacement of a point on the center body, point j in Figure 5-2, is defined as

$$\bar{r}_j = \bar{R} + \bar{\rho}_j \quad (5-5)$$

\bar{r}_j = position vector of point j relative to the Surface Coordinate System.

\bar{R} = position vector of center body center of gravity relative to Surface Coordinate System.

$\bar{\rho}_j$ = position vector of point j relative to the Lander Coordinate System.

Because the center body is experiencing angular velocities and accelerations, the velocity and acceleration of point j are expressed as

$$\dot{\bar{r}}_j = \dot{\bar{R}} + \dot{\bar{\rho}}_j + (\dot{\bar{\Omega}} \times \bar{\rho}_j) \quad (5-6)$$

$$\ddot{\bar{r}}_j = \ddot{\bar{R}} + \ddot{\bar{\rho}}_j + (\ddot{\bar{\Omega}} \times \bar{\rho}_j) + \dot{\bar{\Omega}} \times (\dot{\bar{\Omega}} \times \bar{\rho}_j) + 2(\dot{\bar{\Omega}} \times \dot{\bar{\rho}}_j)$$

In the above, the following definitions apply

- $\dot{\bar{\Omega}}$ = angular velocity of center body in the Lander Coordinate System.
- $\ddot{\bar{\Omega}}$ = angular acceleration of center body in the Lander Coordinate System.
- $\dot{\bar{r}}_j$ = velocity vector of point j relative to the Surface Coordinate System.
- $\ddot{\bar{r}}_j$ = acceleration vector of point j relative to the Surface Coordinate System.
- $\dot{\bar{R}}$ = velocity vector of center body center of gravity relative to the Surface Coordinate System.
- $\ddot{\bar{R}}$ = acceleration vector of center body center of gravity relative to the Surface Coordinate System.
- $\dot{\bar{\rho}}_j$ = velocity vector of point j relative to the Lander Coordinate System.
- $\ddot{\bar{\rho}}_j$ = acceleration vector of point j relative to the Lander Coordinate System.

It is assumed that the position vector locating the point on the center body can be separated into a term which varies with time and a term which remains constant with time. Thus, the location of point j relative to the Lander Coordinate System is

$$\bar{\rho}_j(s_j, t) = \bar{\rho}_{oj}(s_j) + \bar{\rho}_{ej}(s_j, t) \quad (5-7)$$

where

- s_j = coordinates of point j in Lander Coordinate System ($s_j = X_j, Y_j,$ and Z_j).
- $\bar{\rho}_{oj}$ = undeformed position of point j in Lander Coordinate System.
- $\bar{\rho}_{ej}$ = deformed position of point j in Lander Coordinate System measured from the undeformed position of that point.

These position vectors are shown in Figure 5-3.

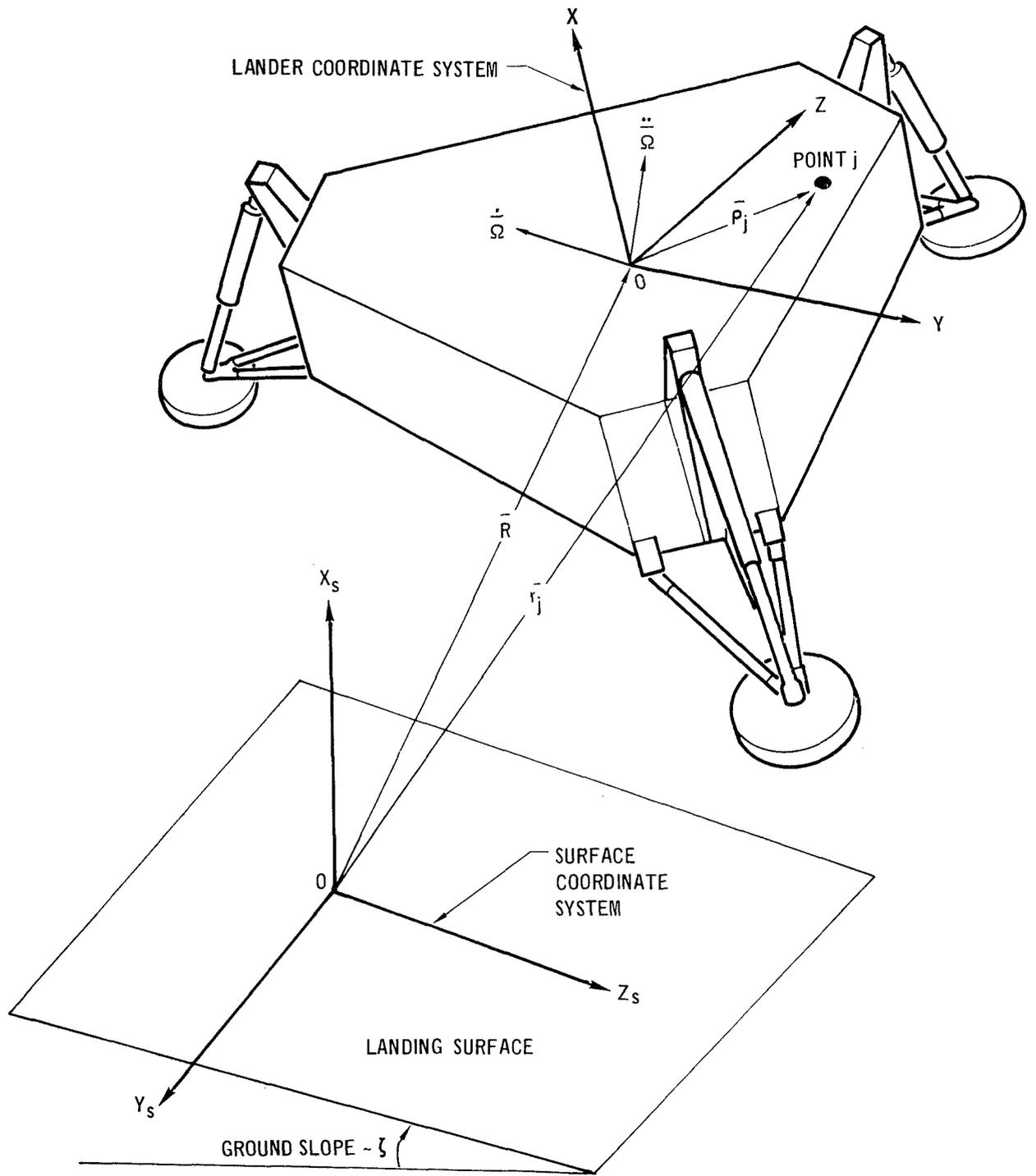


FIGURE 5-2 POSITION OF POINT ON CENTER BODY IN SURFACE COORDINATE SYSTEM

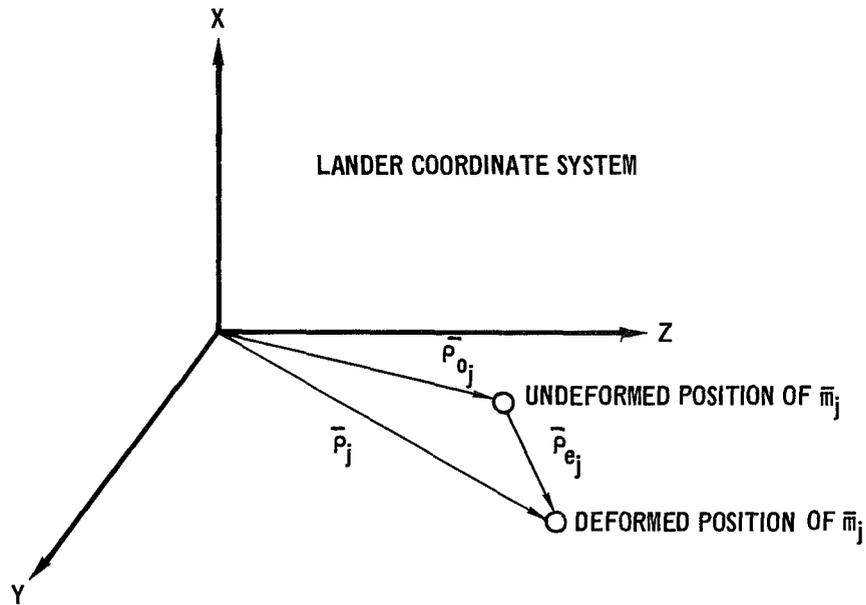


FIGURE 5-3 POSITION OF POINT ON CENTER BODY IN LANDER COORDINATE SYSTEM

Employing the assumption that the elastic deformation is represented by the superposition of a limited number of vibratory modes, the deformed position of point j is

$$\bar{\rho}_{ej} = \sum_n^N \bar{\phi}_{nj} q_n(t) \quad (5-8)$$

where

N = number of modes included.

$\bar{\phi}_{nj}$ = magnitude of n th elastic mode shape at point j . These are a function of the coordinates s_j .

q_n = generalized coordinate associated with n th mode. These are a function of time.

Expressing the deformed position of point j as components in the three axes of the Lander Coordinate System results in the following

$$\bar{\rho}_{ej} = \rho_{ex}^j \bar{i} + \rho_{ey}^j \bar{j} + \rho_{ez}^j \bar{k} \quad (5-9)$$

In the above, \bar{i} , \bar{j} , and \bar{k} are the unit normal vectors in the Lander Coordinate

System. Combining Equations (5-8) and (5-9) results in the following expression for the deformed position of point j.

$$\bar{\rho}_{ej} = \sum_n^N (\phi_{xn}^j \bar{i} + \phi_{yn}^j \bar{j} + \phi_{zn}^j \bar{k}) q_n \quad (5-10)$$

The terms, ϕ_{xn}^j , ϕ_{yn}^j , and ϕ_{zn}^j are the components of the nth mode shape at point j in the three lander axes directions.

The total velocity of point j is obtained by combining Equations (5-6) and (5-7).

$$\dot{\bar{r}}_j = \dot{\bar{R}} + \dot{\bar{\rho}}_{ej} + \dot{\bar{\Omega}} \times (\bar{\rho}_{oj} + \bar{\rho}_{ej}) \quad (5-11)$$

The total kinetic energy of the center body, T, is obtained by summing the kinetic energy of all control points on the center body each having a mass \bar{m}_j

$$T = \frac{1}{2} \sum_j^J \bar{m}_j \dot{\bar{r}}_j \cdot \dot{\bar{r}}_j \quad (5-12)$$

where J equals the total number of mass points on the center body. Combining Equation (5-12) with the definitions given in Equations (5-10) and (5-11), results in the following kinetic energy expression

$$\begin{aligned} T = & \frac{1}{2} M (\dot{q}_x^2 + \dot{q}_y^2 + \dot{q}_z^2) + \frac{1}{2} (I_{xx} \dot{q}_{rx}^2 + I_{yy} \dot{q}_{ry}^2 + I_{zz} \dot{q}_{rz}^2) \\ & - (I_{xy} \dot{q}_{rx} \dot{q}_{ry} + I_{xz} \dot{q}_{rx} \dot{q}_{rz} + I_{yz} \dot{q}_{ry} \dot{q}_{rz}) \\ & + \frac{1}{2} \dot{q}_{rx}^2 \sum_n^N [(P_{yn} + P_{zn}) q_n + \frac{1}{2} N_{xn} q_n^2] \\ & + \frac{1}{2} \dot{q}_{ry}^2 \sum_n^N [(P_{xn} + P_{zn}) q_n + \frac{1}{2} N_{yn} q_n^2] \\ & + \frac{1}{2} \dot{q}_{rz}^2 \sum_n^N [(P_{xn} + P_{yn}) q_n + \frac{1}{2} N_{zn} q_n^2] \\ & + \frac{1}{2} \sum_n^N m_n q_n^2 \end{aligned} \quad (5-13)$$

Terms in Equation (5-13) are defined as follows, where the q_k 's and q_{rk} 's are the generalized coordinates of the center body's rigid body modes.

M = center body mass.

I_{xx}, I_{yy}, I_{zz} = center body moments of inertia

I_{xy}, I_{xz}, I_{yz} = center body products of inertia.

$q_k, \dot{q}_k, \ddot{q}_k$ = rigid body translational displacement, velocity, and acceleration of center body center of gravity in the Surface Coordinate System (for $k = X_s, Y_s,$ or Z_s axes).

$q_{rk}, \dot{q}_{rk}, \ddot{q}_{rk}$ = rigid body angular displacement, velocity, and acceleration relative to Lander Coordinate System (for $k = X, Y,$ or Z).

m_n = generalized mass of nth elastic mode,
 $= \sum_j^J \bar{m}_j \bar{\phi}_{nj} \cdot \bar{\phi}_{nj}$

In addition, the following center body generalized inertia properties are expressed as

$$P_{xn} = \sum_j^J X_j \bar{m}_j \phi_{xn}^j$$

$$P_{yn} = \sum_j^J Y_j \bar{m}_j \phi_{yn}^j$$

$$P_{zn} = \sum_j^J Z_j \bar{m}_j \phi_{zn}^j$$

$$N_{xn} = \sum_j^J \bar{m}_j (\phi_{yn}^j)^2 + \phi_{zn}^j)^2$$

$$N_{yn} = \sum_j^J \bar{m}_j (\phi_{xn}^j)^2 + \phi_{zn}^j)^2$$

$$N_{zn} = \sum_j^J \bar{m}_j (\phi_{xn}^j)^2 + \phi_{yn}^j)^2$$

The potential energy of the center body consists of the potential due to the planet's gravity field and the strain energy due to the center body's elastic deformation.

The potential due to the gravity field is

$$U_g = gM(\cos \zeta q_x - \sin \zeta q_z) \quad (5-14)$$

and the center body's strain energy is

$$U_s = \frac{1}{2} \sum_n^N m_n \omega_n^2 q_n^2 \quad (5-15)$$

where ω_n is the natural frequency at the nth free-free mode. Combining these expressions, the total potential energy of the center body is

$$U = gM(\cos \zeta q_x - \sin \zeta q_z) + \frac{1}{2} \sum_n^N m_n \omega_n^2 q_n^2 \quad (5-16)$$

The final form of the center body's equations of motion were obtained by applying the Lagrangian equations to the energy terms of Equations (5-13) and (5-16). The Lagrangian equations are expressed as

$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{q}_c} \right) - \frac{\partial T}{\partial q_c} + \frac{\partial U}{\partial q_c} = Q_c \quad (5-17)$$

where

q_c, \dot{q}_c = cth generalized coordinate and generalized velocity (either rigid body or elastic modes).

T = center body's kinetic energy.

U = center body's potential energy.

Q_c = generalized force or moment in cth mode.

The generalized forces of the rigid body translational modes are the combination of the inertia effects of noncontacting footpads and the sum of the landing gear strut forces in the respective axis directions. The generalized moments of the rigid body rotational modes are the sum of the moments at the center body center of gravity, about the center body's axes. These consist of

the inertia effects of the noncontacting footpads and moments due to the landing gear strut loads. For the elastic modes, the generalized force in the nth mode is

$$Q_n = \sum_p^P (F_x^p \phi_{nx}^p + F_y^p \phi_{ny}^p + F_z^p \phi_{nz}^p) \quad (5-18)$$

In the above, p refers to the pth point on the center body where a force is applied. There are a total of P forces, each of which has been resolved into components in the Lander Coordinate System axes.

To evaluate the inertia effects of a noncontacting footpad, consider the position vectors shown in Figure 5-4. The position of a footpad in question is given as

$$\bar{r}_{f_i} = \bar{R} + \bar{f}_i \quad (5-19)$$

where

\bar{r}_{f_i} = position vector of ith noncontacting footpad relative to Surface Coordinate System.

\bar{R} = position vector of center body center of gravity relative to Surface Coordinate System.

\bar{f}_i = position vector of ith noncontacting footpad relative to Lander Coordinate System.

With the assumption that the noncontacting footpad is on an extension of the center body structure, its acceleration, \ddot{r}_{f_i} , is expressed as

$$\ddot{r}_{f_i} = \ddot{R} + (\ddot{\Omega} \times \bar{f}) + \dot{\Omega} \times (\dot{\Omega} \times \bar{f}) \quad (5-20)$$

In the above, the terms \ddot{R} , $\ddot{\Omega}$, and $\dot{\Omega}$ are defined in Figure 5-2 and Equation (5-6). Thus, the total inertia loads of all the noncontacting footpads at the center body center of gravity are

$$\bar{F}_f = - \sum_i^I m_i \ddot{r}_{f_i} \quad (5-21)$$

and

$$\bar{T}_f = - \sum_i^I m_i (\bar{f}_i \times \ddot{r}_{f_i})$$

where I is the total number of noncontacting footpads, each with a mass m_i .

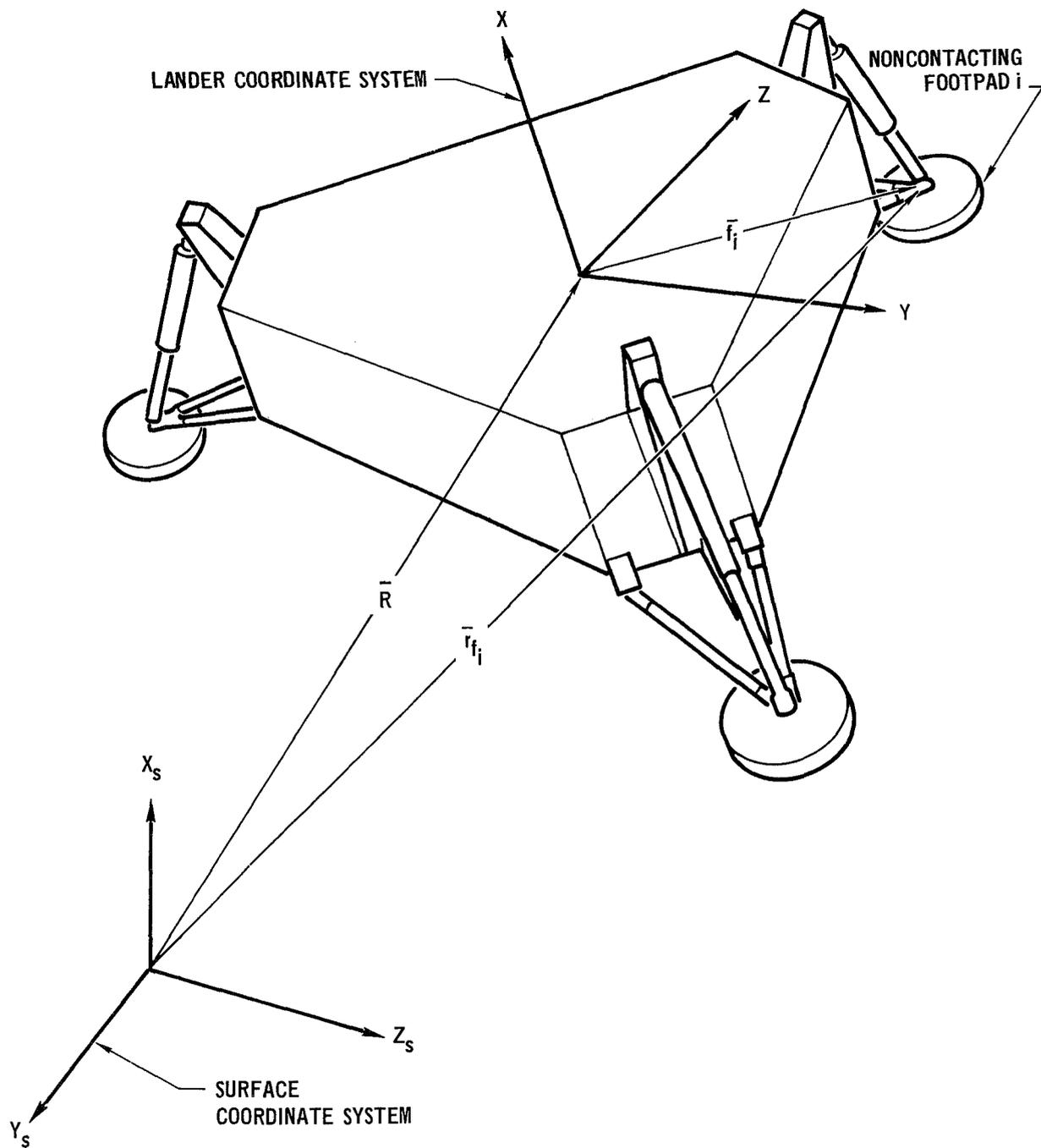


FIGURE 5-4 POSITION OF NONCONTACTING FOOTPAD IN SURFACE COORDINATE SYSTEM

Equations (5-21) may be written as

$$\begin{pmatrix} F_{xf} \\ F_{yf} \\ F_{zf} \end{pmatrix} = - \sum_i^I m_i \begin{pmatrix} \ddot{q}_x \\ \ddot{q}_y \\ \ddot{q}_z \end{pmatrix} - \sum_i^I m_i [DC] \begin{bmatrix} 0 & -f_{z_i} & f_{y_i} \\ f_{z_i} & 0 & -f_{x_i} \\ -f_{y_i} & f_{x_i} & 0 \end{bmatrix} \begin{pmatrix} \dot{q}_{rx} \\ \dot{q}_{ry} \\ \dot{q}_{rz} \end{pmatrix}$$

(5-22)

$$- \sum_i^I m_i [DC] \begin{pmatrix} \dot{q}_{ry} (\dot{q}_{rx} f_{y_i} - \dot{q}_{ry} f_{x_i}) - \dot{q}_{rz} (\dot{q}_{rz} f_{x_i} - \dot{q}_{rx} f_{z_i}) \\ \dot{q}_{rz} (\dot{q}_{ry} f_{z_i} - \dot{q}_{rz} f_{y_i}) - \dot{q}_{rx} (\dot{q}_{rx} f_{y_i} - \dot{q}_{ry} f_{x_i}) \\ \dot{q}_{rx} (\dot{q}_{rz} f_{x_i} - \dot{q}_{rx} f_{z_i}) - \dot{q}_{ry} (\dot{q}_{ry} f_{z_i} - \dot{q}_{rz} f_{y_i}) \end{pmatrix}$$

and

$$\begin{pmatrix} T_{xf} \\ T_{yf} \\ T_{zf} \end{pmatrix} = - \sum_i^I m_i \begin{bmatrix} (f_{y_i}^2 + f_{z_i}^2) & -f_{x_i} f_{y_i} & -f_{x_i} f_{z_i} \\ -f_{x_i} f_{y_i} & (f_{x_i}^2 + f_{z_i}^2) & -f_{y_i} f_{z_i} \\ -f_{x_i} f_{z_i} & -f_{y_i} f_{z_i} & (f_{x_i}^2 + f_{y_i}^2) \end{bmatrix} \begin{pmatrix} \dot{q}_{rx} \\ \dot{q}_{ry} \\ \dot{q}_{rz} \end{pmatrix}$$

$$- \sum_i^I m_i \begin{bmatrix} 0 & \tilde{f}_{z_i} & \tilde{f}_{y_i} \\ \tilde{f}_{z_i} & 0 & -\tilde{f}_{x_i} \\ -\tilde{f}_{y_i} & \tilde{f}_{x_i} & 0 \end{bmatrix} \begin{pmatrix} \ddot{q}_x \\ \ddot{q}_y \\ \ddot{q}_z \end{pmatrix}$$

(5-22) (Continued)

$$- \sum_i^I m_i \begin{pmatrix} f_{x_i} \dot{q}_{rx} (f_{y_i} \dot{q}_{rz} - f_{z_i} \dot{q}_{ry}) + (f_{y_i}^2 - f_{z_i}^2) \dot{q}_{ry} \dot{q}_{rz} + f_{y_i} f_{z_i} (\dot{q}_{rz}^2 - \dot{q}_{ry}^2) \\ f_{y_i} \dot{q}_{ry} (f_{z_i} \dot{q}_{rx} - f_{x_i} \dot{q}_{rz}) + (f_{z_i}^2 - f_{x_i}^2) \dot{q}_{rx} \dot{q}_{rz} + f_{x_i} f_{z_i} (\dot{q}_{rx}^2 - \dot{q}_{rz}^2) \\ f_{z_i} \dot{q}_{rz} (f_{x_i} \dot{q}_{ry} - f_{y_i} \dot{q}_{rx}) + (f_{x_i}^2 - f_{y_i}^2) \dot{q}_{rx} \dot{q}_{ry} + f_{x_i} f_{y_i} (\dot{q}_{ry}^2 - \dot{q}_{rx}^2) \end{pmatrix}$$

where

$$\begin{pmatrix} \tilde{f}_{x_i} \\ \tilde{f}_{y_i} \\ \tilde{f}_{z_i} \end{pmatrix} = [DC] \begin{pmatrix} f_{x_i} \\ f_{y_i} \\ f_{z_i} \end{pmatrix}$$

are the components of the i th footpad location, from the center body center of gravity, in the Surface Coordinate System and

$f_{x_i}, f_{y_i}, f_{z_i}$ = components of position vector locating i th noncontacting footpad in the Lander Coordinate System

[DC] = direction cosine matrix, Equation (5-1), relating Lander Coordinate System to Surface Coordinate System.

F_{xf}, F_{yf}, F_{zf} = inertia forces due to noncontacting footpads, relative to Surface Coordinate System.

T_{xf}, T_{yf}, T_{zf} = inertia moments due to noncontacting footpad, relative to Lander Coordinate System.

Using the center body energy expressions, Equations (5-13) and (5-16), and the noncontacting footpad inertia loads, Equation (5-22), in conjunction with the Lagrangian equation leads to the center body equations of motion given in Equation (5-23).

5.1.3 Landing Gear Strut Idealization - The two landing gear configuration options available in the Landing Loads and Motions Program are applicable to the inverted tripod gear and the cantilever gear. Each landing gear consists of a main strut and two drag struts. It is assumed that these are pinned struts and thus moments or torsion are not introduced at their ends. Both the main strut and the drag struts are capable of carrying tension and compression loads and may possess either velocity dependent energy absorption characteristics, stroke dependent characteristics, or a combination of the two. The energy absorption characteristics of all main struts in a given lander are the same. Similarly, characteristics of all the drag struts are the same, however they may be different than those of the main struts. For a cantilever gear, the effect of bending in the main strut is included by altering the energy absorption properties of the drag struts.

Relative motion between each footpad and the center body is used to determine the stroke and velocity of stroke in each strut. This information is used to determine the load in a landing gear strut employing the subroutine STRUT.

5.1.3.1 Stroke Dependent Attenuation - The primary type of energy absorption mechanism for the landing gear system consists of crushable

$$\begin{bmatrix}
 M + \sum \frac{1}{2} m_i & 0 & 0 \\
 0 & M + \sum \frac{1}{2} m_i & 0 \\
 0 & 0 & M + \sum \frac{1}{2} m_i \\
 0 & -\sum \frac{1}{2} m_i f_{x_i} z_i & \sum \frac{1}{2} m_i f_{x_i} y_i \\
 \sum \frac{1}{2} m_i f_{x_i} z_i & 0 & -\sum \frac{1}{2} m_i f_{x_i} y_i \\
 -\sum \frac{1}{2} m_i f_{x_i} y_i & \sum \frac{1}{2} m_i f_{x_i} z_i & 0
 \end{bmatrix}
 \begin{bmatrix}
 0 \\
 \sum \frac{1}{2} m_i f_{z_i} \\
 -\sum \frac{1}{2} m_i f_{y_i} \\
 -\sum \frac{1}{2} m_i f_{z_i} z_i \\
 0 \\
 0
 \end{bmatrix}
 +
 \begin{bmatrix}
 -\sum \frac{1}{2} m_i f_{z_i} z_i & \sum \frac{1}{2} m_i f_{y_i} z_i \\
 \sum \frac{1}{2} m_i f_{z_i} z_i & -\sum \frac{1}{2} m_i f_{y_i} z_i \\
 -\sum \frac{1}{2} m_i f_{y_i} z_i & \sum \frac{1}{2} m_i f_{z_i} z_i \\
 -\sum \frac{1}{2} m_i f_{z_i} z_i & -\sum \frac{1}{2} m_i f_{y_i} z_i \\
 -\sum \frac{1}{2} m_i f_{y_i} z_i & \sum \frac{1}{2} m_i f_{z_i} z_i \\
 -\sum \frac{1}{2} m_i f_{z_i} z_i & -\sum \frac{1}{2} m_i f_{y_i} z_i
 \end{bmatrix}
 \begin{bmatrix}
 \ddot{q}_x \\
 \ddot{q}_y \\
 \ddot{q}_z \\
 \ddot{q}_{rx} \\
 \ddot{q}_{ry} \\
 \ddot{q}_{rz}
 \end{bmatrix}
 =
 \begin{bmatrix}
 F_x - g M \cos \zeta \\
 F_y \\
 F_z + g M \sin \zeta \\
 T_x \\
 T_y \\
 T_z
 \end{bmatrix}$$

$$\begin{bmatrix}
 \sum \frac{1}{2} m_i \ddot{q}_{ry} (\ddot{q}_{rx} f_{y_i} - \ddot{q}_{ry} f_{x_i}) - \ddot{q}_{rz} (\ddot{q}_{rx} f_{z_i} - \ddot{q}_{ry} f_{z_i}) \\
 -\sum \frac{1}{2} m_i \ddot{q}_{rz} (\ddot{q}_{ry} z_i - \ddot{q}_{rz} f_{y_i}) - \ddot{q}_{rx} (\ddot{q}_{rx} f_{y_i} - \ddot{q}_{ry} f_{x_i}) \\
 \sum \frac{1}{2} m_i \ddot{q}_{rx} (\ddot{q}_{rz} f_{x_i} - \ddot{q}_{rx} z_i) - \ddot{q}_{ry} (\ddot{q}_{ry} z_i - \ddot{q}_{rz} f_{y_i}) \\
 \ddot{q}_{rx} \ddot{q}_{rz} (lx + \sum \frac{1}{2} m_i f_{x_i} f_{y_i}) - \ddot{q}_{rx} \ddot{q}_{ry} (lyz + \sum \frac{1}{2} m_i f_{y_i} z_i) + \ddot{q}_{ry} \ddot{q}_{rz} (lyz + \sum \frac{1}{2} m_i f_{x_i} f_{z_i}) \\
 \ddot{q}_{rx} \ddot{q}_{ry} (lyz + \sum \frac{1}{2} m_i f_{y_i} z_i) - \ddot{q}_{ry} \ddot{q}_{rz} (lyz + \sum \frac{1}{2} m_i f_{x_i} f_{z_i}) + \ddot{q}_{rx} \ddot{q}_{rz} (lx + \sum \frac{1}{2} m_i f_{x_i} f_{z_i}) \\
 \ddot{q}_{ry} \ddot{q}_{rz} (lyz + \sum \frac{1}{2} m_i f_{x_i} f_{z_i}) - \ddot{q}_{rx} \ddot{q}_{rz} (lyz + \sum \frac{1}{2} m_i f_{y_i} z_i) + \ddot{q}_{ry} \ddot{q}_{rz} (lyz + \sum \frac{1}{2} m_i f_{x_i} f_{z_i})
 \end{bmatrix}
 -
 \begin{bmatrix}
 \sum \frac{1}{2} m_i \ddot{q}_{ry} (\ddot{q}_{rx} f_{y_i} - \ddot{q}_{ry} f_{x_i}) - \ddot{q}_{rz} (\ddot{q}_{rx} f_{z_i} - \ddot{q}_{ry} f_{z_i}) \\
 \sum \frac{1}{2} m_i \ddot{q}_{rz} (\ddot{q}_{ry} z_i - \ddot{q}_{rz} f_{y_i}) - \ddot{q}_{rx} (\ddot{q}_{rx} f_{y_i} - \ddot{q}_{ry} f_{x_i}) \\
 \sum \frac{1}{2} m_i \ddot{q}_{rx} (\ddot{q}_{rz} f_{x_i} - \ddot{q}_{rx} z_i) - \ddot{q}_{ry} (\ddot{q}_{ry} z_i - \ddot{q}_{rz} f_{y_i}) \\
 \sum \frac{1}{2} m_i \ddot{q}_{ry} (\ddot{q}_{rx} f_{y_i} - \ddot{q}_{ry} f_{x_i}) - \ddot{q}_{rz} (\ddot{q}_{rx} f_{z_i} - \ddot{q}_{ry} f_{z_i}) \\
 \sum \frac{1}{2} m_i \ddot{q}_{rz} (\ddot{q}_{ry} z_i - \ddot{q}_{rz} f_{y_i}) - \ddot{q}_{rx} (\ddot{q}_{rx} f_{y_i} - \ddot{q}_{ry} f_{x_i}) \\
 \sum \frac{1}{2} m_i \ddot{q}_{rx} (\ddot{q}_{rz} f_{x_i} - \ddot{q}_{rx} z_i) - \ddot{q}_{ry} (\ddot{q}_{ry} z_i - \ddot{q}_{rz} f_{y_i})
 \end{bmatrix}$$

(5-23)

AND

$$m_n \ddot{q}_n + \omega_n^2 m_n q_n = Q_n + \ddot{q}_{rx}^2 (P_{yn} + P_{zn}) + N_{xn} q_n + \ddot{q}_{ry}^2 (P_{xn} + P_{zn}) + N_{yn} q_n + \ddot{q}_{rz}^2 (P_{xn} + P_{yn}) + N_{zn} q_n \quad \text{FOR } n=1, 2, \dots, N$$

cartridges housed inside the landing gear struts. Each cartridge crushes at a constant load when the landing gear strut is stroked. Several cartridges, possessing different crushing strengths, may be stacked in series to form a desired load-stroke characteristic. In the present study, all struts may contain a maximum of five cartridges to attenuate compression loads and five to attenuate tension loads.

The idealization of the crushable landing gear struts in the Landing Loads and Motions Program is the same as is employed in the Landing Gear Option of the Structural Analysis Program. A discussion explaining the procedure for determining strut loads during strut stroking is presented in Section 4.1.2.2. In addition, a typical load-stroke relationship for a strut is shown in Figure 4-18.

5.1.3.2 Main Strut Bending - For a cantilever gear, the drag strut loads acting normal to the main strut in combination with the lateral loads on the footpad cause lateral deflections of the main strut. The assumption is made that the effect of this main strut deflection can be approximated through a modification of the drag strut load-stroke relationship.

The main strut is idealized as a simply supported beam whose elastic axis is defined by simple beam theory. Neglecting the effect of axial loads on the lateral deflection of the main strut, the deflected main strut is shown in Figure 5-5. The force F_n is the component of the drag strut force normal to the main strut axis. This force is defined as

$$F_n \bar{n} = F(\bar{s} \cdot \bar{n})\bar{n} \quad (5-24)$$

where F is the force in the drag strut, \bar{n} is a unit vector normal to the main strut, and \bar{s} is a unit vector in the direction of the drag strut. The flexural stiffness of the main strut on either side of the drag strut attach point is defined as EI_1 and EI_2 .

The lateral deflection of the main strut due to the load F_n is expressed as

$$\Delta \bar{n} = F_n \left[\frac{1}{EI_1} \left(\frac{a^3 b}{3l} - \frac{a^4 b}{3l^2} \right) + \frac{1}{EI_2} \left(\frac{a^4}{l} - \frac{a^5}{3l^2} + \frac{a^2 l}{3} - a^3 \right) \right] \bar{n} \quad (5-25)$$

The magnitude of this deflection, λ , in the direction of the drag strut

axis is

$$\lambda \bar{s} = \Delta(\bar{s} \cdot \bar{n}) \bar{s} \quad (5-26)$$

A spring constant reflecting the bending deflection of the main strut is defined as

$$K_B = \frac{F}{\lambda} \quad (5-27)$$

This expression is evaluated with the aid of Equations (5-24), (5-25) and (5-26) and results in

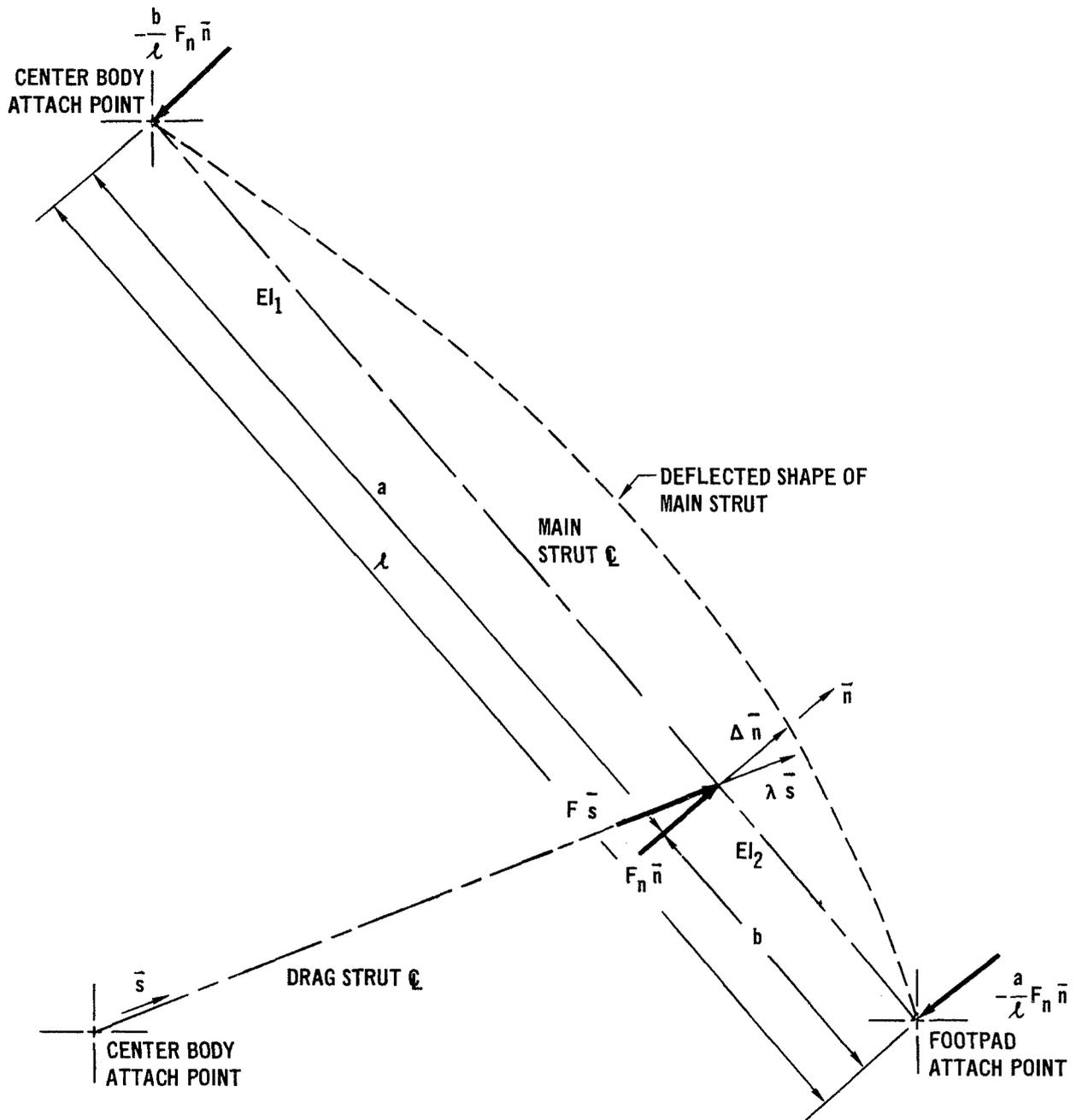
$$K_B = \frac{1}{(\bar{s} \cdot \bar{n})^2 \left[\frac{1}{EI_1} \left(\frac{a^3 b}{3\ell} - \frac{a^4 b}{3\ell^2} \right) + \frac{1}{EI_2} \left(\frac{a^4}{\ell} - \frac{a^5}{3\ell^2} + \frac{a^2 \ell}{3} - a^3 \right) \right]} \quad (5-28)$$

This linear spring is assumed to be in series with the spring defining each elastic portion of the load-stroke diagram of the drag strut. A modified spring constant for these elastic portions of the load-stroke curve is calculated to reflect the main strut bending. This modified spring constant is used to define a modified load-stroke relationship for the drag strut. A typical drag strut modified load-stroke curve is shown in Figure 5-6. This modified curve is then employed to define the loads in the drag strut.

During stroking of the main strut, its bending characteristics change due to the change in strut length. These changes in bending characteristics are incorporated by continually modifying the spring constant of the drag strut load-stroke curve. As a result the drag strut load-stroke relationship is continually updated to reflect these changes in main strut bending characteristics.

5.1.3.3 Velocity Dependent Attenuation - Provisions are available for the inclusion of a constant friction force and a velocity dependent damping force in each strut. Both of these are applied in a direction opposite to the velocity of the stroking motion in the strut. The combination of these force terms is expressed as

$$F_v = - \frac{\dot{S}}{|\dot{S}|} [F_r + C_v |\dot{S}|^Y] \quad (5-29)$$



FOLLOWING RELATIONSHIPS HOLD:

$$F_n \vec{n} = F(\vec{s} \cdot \vec{n}) \vec{n}$$

$$\lambda \vec{s} = \Delta (\vec{n} \cdot \vec{s}) \vec{s}$$

FIGURE 5-5 DEFLECTION OF CANTILEVER GEAR MAIN STRUT

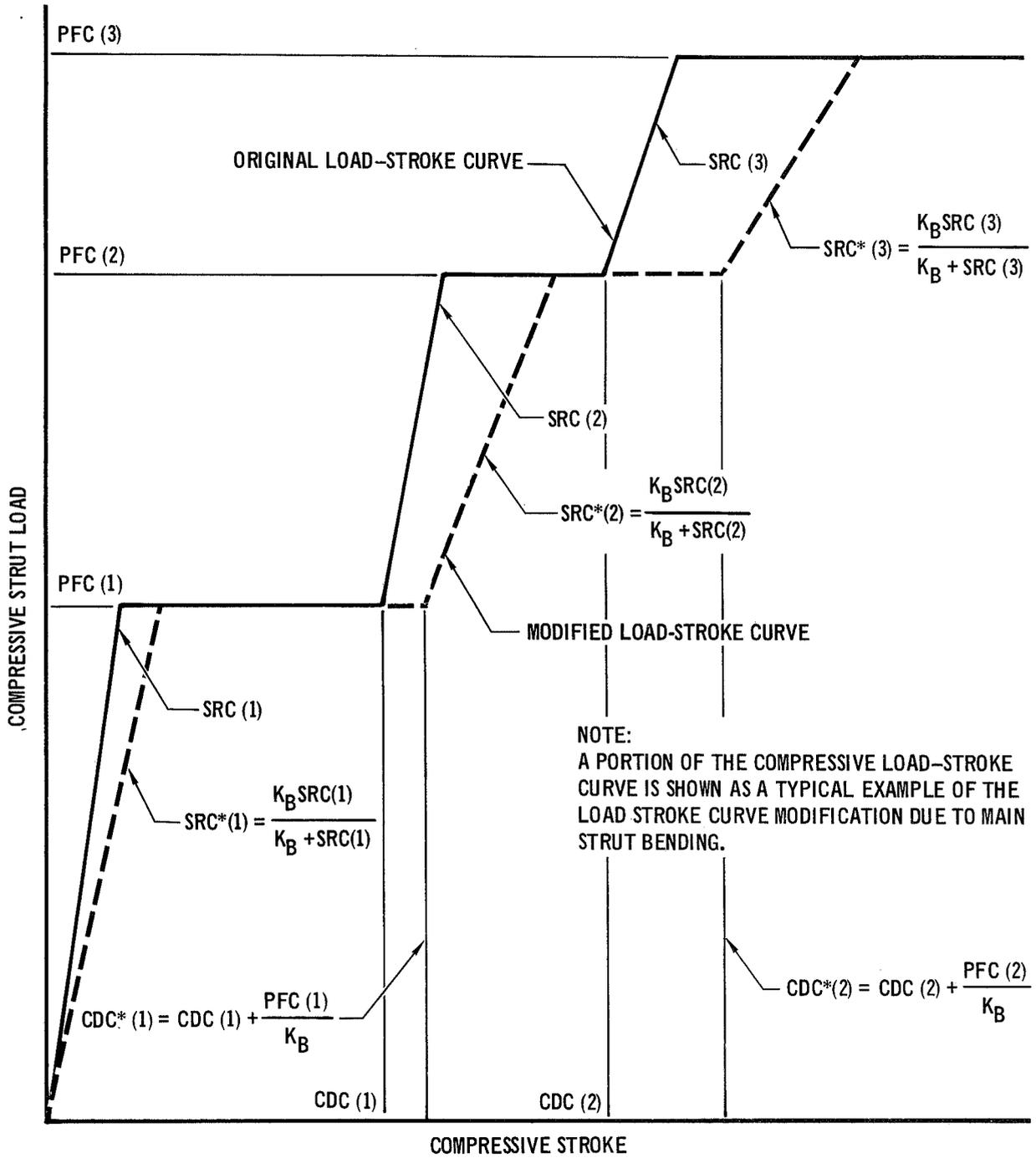


FIGURE 5-6 MODIFIED DRAG STRUT LOAD-STROKE CURVE

where

F_v = total velocity dependent force along axis of strut.

\dot{S} = stroking velocity of strut.

F_r = magnitude of constant friction force (input quantity).

C_v = coefficient of velocity proportional force (input quantity).

γ = power of velocity in velocity proportional force (input quantity).

This force is superimposed on the crushing force or may be included by itself.

5.1.4 Soil Mechanics - Two methods of representing the footpad-soil interaction are incorporated in the Landing Loads and Motions Program. The first of these, referred to as the Primary Soil Mechanics Method, is a modification of the soil mechanics analyses developed during the Lunar Module (LM) soil mechanics studies. This modification is similar to that employed during the Surveyor analysis. An alternate soil mechanics method, referred to as the Secondary Soil Mechanics Method, determines the soil force through a simple elastic-plastic relationship between soil pressure and depth of soil penetration. On an optional basis, a crushable footpad attenuation system located on the bottom of the footpads may be included with either of these soil mechanics routines. The footpad attenuation system is discussed in Section 5.1.5. Symbols employed in the soil mechanics routines are defined in Figure 5-7. The soil forces are determined in the subroutine SOIL.

The surface of each footpad is represented by a number of concentric conical and/or cylindrical segments as shown in Figure 5-8. It is assumed that the footpad is always aligned with the plane of the landing surface. Upon entering the soil mechanics routine, the footpad velocities and soil penetration indicated in Figure 5-8 are available. The three components of the soil force in the Footpad Coordinate System are returned from the soil mechanics routine.

5.1.4.1 Primary Soil Mechanics - The footpad-soil interaction method developed in Reference (11) for the LM shaped footpad utilized principles which are fundamental to the interaction phenomenon occurring during soil penetration. Applicability of this basic method to a different footpad shape was demonstrated by the good agreement obtained between telemetered Surveyor lunar impact data and predicted landing dynamics as reported in Reference (12).

SYMBOL	DEFINITION	UNITS*
A	FOOTPAD AREA PROJECTED ON LANDING SURFACE	L ²
A _p	FOOTPAD AREA PROJECTED ON PLANE PARALLEL TO PLANE DEFINED BY SURFACE NORMAL AND VELOCITY V _{ap}	L ²
A _⊥	FOOTPAD AREA PROJECTED ON PLANE NORMAL TO VELOCITY V _{ap}	L ²
C _{ms}	SOIL DYNAMIC MECHANICAL STRENGTH COEFFICIENT	--
C _d	SOIL DRAG COEFFICIENT	--
d	DEPTH OF SOIL PENETRATION	L
dr/dd	CHANGE IN FOOTPAD RADIUS WITH RESPECT TO SOIL PENETRATION	--
D _r	RELATIVE DENSITY OF SOIL (NO COMPACTION, D _r = 0; MAXIMUM COMPACTION, D _r = 1)	--
F _{ap}	SOIL FORCE PARALLEL TO FOOTPAD VELOCITY	F
F _{np}	SOIL FORCE NORMAL TO FOOTPAD VELOCITY	F
F _x , F _y , F _z	COMPONENTS OF SOIL FORCE IN FOOTPAD COORDINATE SYSTEM	F
g	LOCAL ACCELERATION OF GRAVITY	L/T ²
g _e	EARTH ACCELERATION OF GRAVITY	L/T ²
r	FOOTPAD RADIUS AT FOOTPAD-LANDING SURFACE INTERSECTION	L
r _m	MAXIMUM FOOTPAD RADIUS	L
R ₁ , R ₂ , R ₃ , R ₄	PARAMETERS DEFINING FOOTPAD SHAPE (SEE FIGURE 5-8)	L
S ₁ , S ₂ , S ₃ , S ₄	PARAMETERS DEFINING FOOTPAD SHAPE (SEE FIGURE 5-8)	L
V _{ap}	TOTAL VELOCITY OF FOOTPAD	L/T
V _x , V _y , V _z	COMPONENTS OF FOOTPAD VELOCITY IN FOOTPAD COORDINATE SYSTEM	L/T
γ	SOIL UNIT WEIGHT	F/L ³
η	WEDGE SHAPE FACTOR FOR MOVING SOIL MASS	--
⊖	ANGLE DEFINING DIRECTION OF VELOCITY V _{ap} RELATIVE TO SURFACE NORMAL	A
λ	RATIO OF AXIAL TO NORMAL SOIL FORCE	--
ρ	BULK MASS DENSITY OF SOIL	FT ² /L ⁴
Φ	INTERNAL FRICTION ANGLE OF SOIL	A

* UNITS:

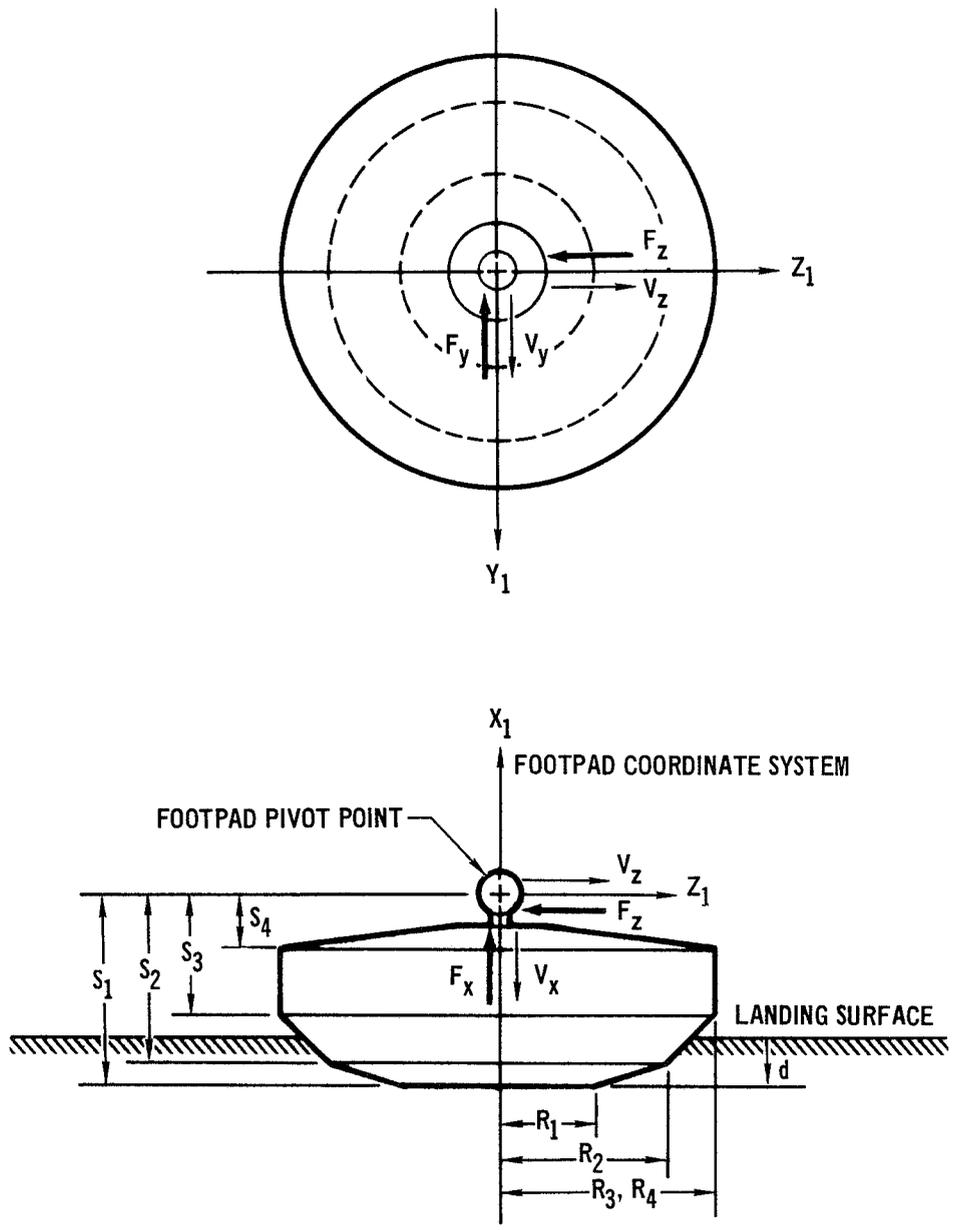
L - LENGTH

F - FORCE

T - TIME

A - ANGLE

FIGURE 5-7 SYMBOLS FOR SOIL MECHANICS ROUTINE



d = DEPTH OF SOIL PENETRATION
 V_x, V_y, V_z = VELOCITIES OF FOOTPAD IN FOOTPAD COORDINATE SYSTEM
 F_x, F_y, F_z = COMPONENTS OF SOIL FORCE IN FOOTPAD COORDINATE SYSTEM

FIGURE 5-8 FOOTPAD REPRESENTATION

In Reference (11) a theory of soil elasto-plastic deformation is used to define the force between the footpad and the deformed soil surface. The soil mass, displaced by the moving footpad, is considered as a degree of freedom independent of the lander system. A spring, representing the soil elasticity, is placed between the footpad and the soil mass. Additional external forces applied to the soil mass represent a momentum transfer force and a force due to the soil strength.

It was shown in the Surveyor simulation that sufficient accuracy can be obtained by neglecting soil elasticity and assuming that the moving soil mass is attached rigidly to the footpad. This simplification results in the removal of the soil mass differential equation from the analysis. Therefore, the soil force acting on the footpad is considered to be the sum of a soil strength term, a soil drag term, and a term approximating the effect of the changing soil mass.

The Primary Soil Mechanics Method employs the empirical relationships discussed in Reference (12). Forces acting on each footpad consist of an axial force, F_{ap} , parallel to the velocity vector of the footpad and a force, F_{np} , normal to the velocity vector.

The axial force is the sum of the forces due to the soil strength, soil drag, and effect of the changing soil mass and is expressed as

$$F_{ap} = C_{ms} \rho g d A_{\theta} + C_d \rho A_{\theta} V_{ap}^2 + 3\eta \rho r^2 \frac{dr}{dd} \left(\frac{A_{\theta}}{A} \right)^{3/2} V_{ap}^2 \cos \theta \quad (5-30)$$

Coefficients C_{ms} and C_d , representing the soil dynamic mechanical strength and drag coefficients respectively, are empirical factors determined from test and discussed further in later paragraphs. The wedge shape factor for moving soil mass, η , is a function of angle of internal friction of the soil, and is also discussed further in the section on empirical relationships. The term $(A_{\theta}/A)^{3/2}$ accounts for the effect of the angle θ . Change in footpad radius with respect to depth of the footpad, dr/dd , is defined by footpad geometry at the current depth of soil penetration.

The force acting on a footpad normal to the velocity vector is

$$F_{np} = \lambda F_{ap} \quad (5-31)$$

This force is always directed out of the soil and is in the plane defined by

the surface normal and the velocity vector. The quantity λ is discussed in the paragraphs defining the empirical relationships. Figure 5-9 presents a diagrammatic representation of these two soil forces.

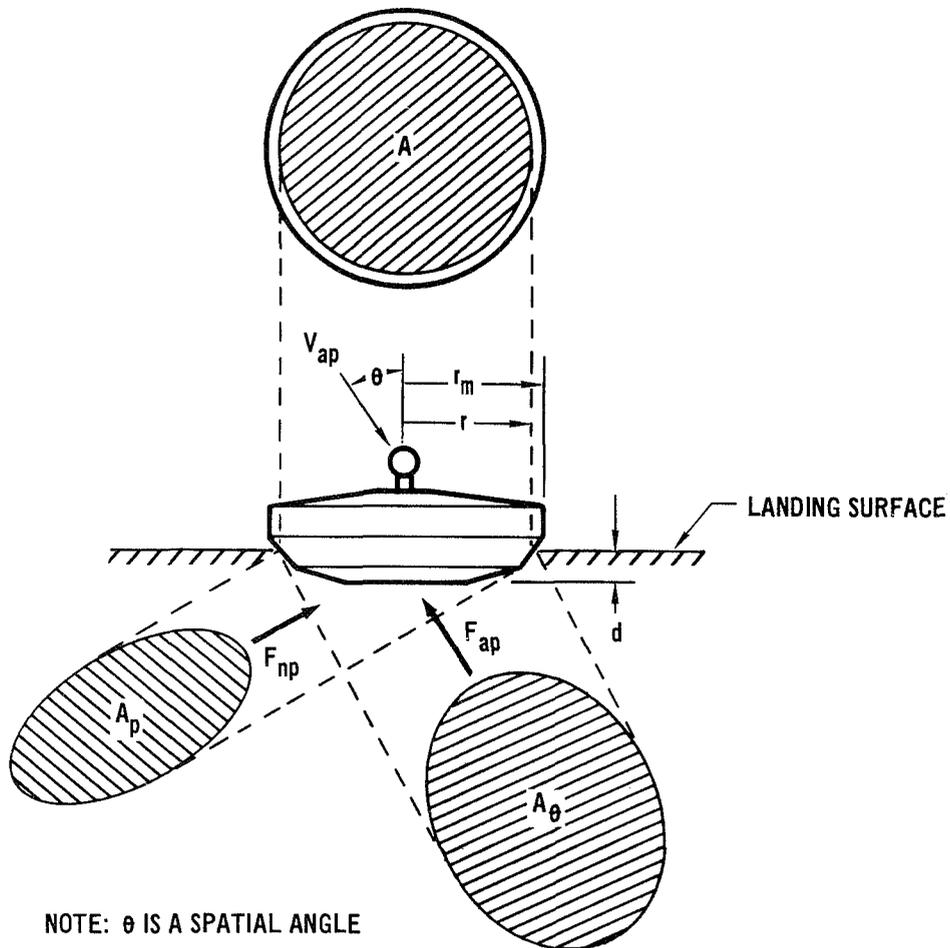


FIGURE 5-9 SOIL FORCES PRIMARY SOIL MECHANICS METHOD

Empirical relationships are determined in terms of the soil properties from impact and drag tests conducted during the LM study. Properties of twelve soils are described in Figure 5-10. The most significant of these properties are: unit weight, γ (bulk mass density, ρ , times g_e); relative density, D_r ; the angle of internal friction, ϕ ; and, to a lesser degree, the

elastic modulus of the soil, E. Based on results reported in Reference (12), the properties γ , D_r and ϕ are adequate to describe the soil for landing dynamics and are used in the present study. In the soil mechanics subroutine, the empirical relationships for C_{ms} , C_d , λ , and η used for the Surveyor footpad, are employed during the determination of the soil forces acting on each footpad. These relationships are presented in Figure 5-11.

NO.	BENDIX DESIGNATION	DESCRIPTION	RELATIVE DENSITY D_r	UNIT WEIGHT γ dynes/cm ³	FRICTION ANGLE ϕ deg.	ELASTIC MODULUS E dynes/cm ²
1	RS LOOSE	RED CRUSHED VOLCANIC SCORIA (NARROWLY GRADED)	0	650	40	379×10^6
2	PS LOOSE	WHITE CRUSHED PUMICE (NARROWLY GRADED)	0	374	43	310×10^6
3	RS INTER	RED CRUSHED VOLCANIC SCORIA (NARROWLY GRADED)	.45	721	44.5	593×10^6
4	RS DENSE	RED CRUSHED VOLCANIC SCORIA(NARROWLY GRADED)	.80	795	47.3	752×10^6
5	RSM-a LOOSE	MIXTURE OF RS AND CRUSHED MARBLE (MS) (NARROWLY GRADED)	0	914	37	538×10^6
6	RC2 LOOSE	RED CRUSHED VOLCANIC SCORIA(BROADLY GRADED)	0	965	43	552×10^6
7	SS LOOSE	WHITE SILICA SAND (WEDRON 40 40- NARROWLY GRADED)	0	1488	29	690×10^6
8	SS INTER.	WHITE SILICA SAND (WEDRON 40 40- NARROWLY GRADED)	.53	1634	36.8	1241×10^6
9	SS DENSE	WHITE SILICA SAND (WEDRON 40 40- NARROWLY GRADED)	.69	1681	39	1655×10^6
10	LSM INTER.	MIXTURE OF RC AND AIR-FLOATED CLAY	.50	1217	42	276×10^6
11	LSM DENSE	MIXTURE OF RC AND AIR-FLOATED CLAY	.70	1288	42	414×10^6
12	RSM-b DENSE	MIXTURE OF RS AND CRUSHED MARBLE (MS) (NARROWLY GRADED)	.75	1335	48	1380×10^6

*TO OBTAIN BULK MASS DENSITY OF SOIL, DIVIDE THIS QUANTITY BY EARTH ACCELERATION OF GRAVITY.
THIS INFORMATION OBTAINED FROM REFERENCE (11).

FIGURE 5-10 PROPERTIES OF SOILS

WEDGE SHAPE FACTOR FOR MOVING SOIL MASS:

$$\eta = \frac{\pi}{3} \frac{1 + \tan \phi/2}{1 - \tan \phi/2}$$

SOIL DYNAMIC MECHANICAL STRENGTH COEFFICIENT:

$$C_{ms} = 29 e^{1.4 D_r} \tan \phi$$

SOIL DRAG COEFFICIENT:

$$C_d = 0.8 + \left(\frac{g}{g_e}\right) (4 + 80 D_r) \left(\frac{r}{r_m}\right)^2 f(\theta) \tan \phi \quad \text{FOR } D_r < 0.5$$

$$C_d = 0.8 + 4 \left(\frac{g}{g_e}\right) e^{4.83 D_r} \left(\frac{r}{r_m}\right)^2 f(\theta) \tan \phi \quad \text{FOR } D_r \geq 0.5$$

WHERE:

$$f(\theta) = 1 - \frac{2\theta}{\pi} \quad \text{FOR } 0 \leq \theta < 45^\circ$$

$$f(\theta) = 0 \quad \text{FOR } \theta \geq 45^\circ$$

RATIO OF AXIAL TO NORMAL SOIL FORCE:

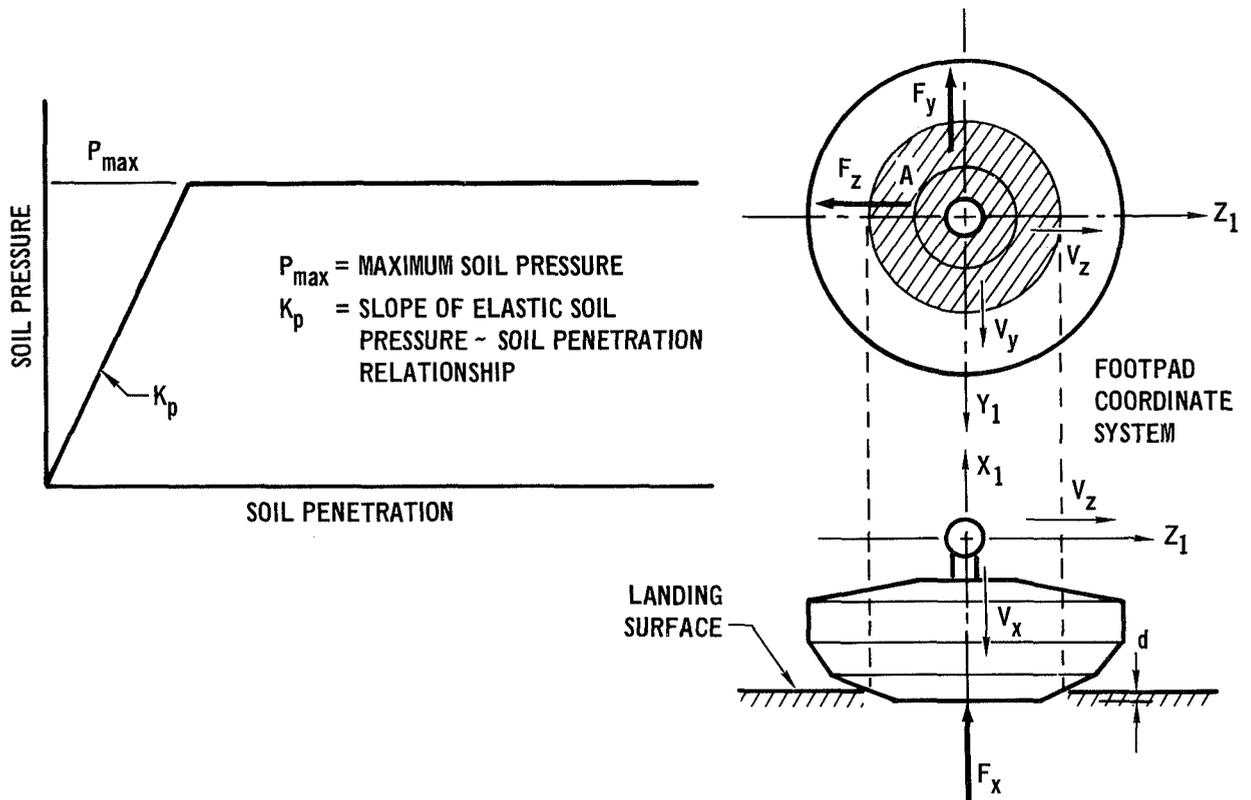
$$\lambda = 0.25 \left(\frac{A_p}{A_\theta}\right) (1 - e^{-50\theta}) (1 + \sin \theta) \quad \text{FOR } 0 \leq \theta < 90^\circ$$

$$\lambda = 0.5 \left(\frac{A_p}{A_\theta}\right) \quad \text{FOR } 90^\circ \leq \theta < 135^\circ$$

$$\lambda = 0 \quad \text{FOR } \theta \geq 135^\circ$$

FIGURE 5-11 EMPIRICAL SOIL RELATIONSHIPS

5.1.4.2 Secondary Soil Mechanics - An alternate soil mechanics routine is available in the Landing Loads and Motions Program. This method determines the pressure acting on a footpad in terms of depth of penetration of the footpad. The pressure-penetration relationship is defined as shown in Figure 5-12. Initially the soil pressure increases linearly from zero at zero penetration to a selected pressure at a specified cutoff depth. Beyond this depth, the pressure remains constant. The normal soil force is the product of the pressure determined from the relationship shown in Figure 5-12 and the area of the footpad projected on the landing surface.



THE SOIL FORCE NORMAL TO THE LANDING SURFACE OBTAINED BY THE SECONDARY SOIL MECHANICS METHOD IS GIVEN AS

$$F_x = A d K_p \text{ OR } F_x = A P_{max}$$

WHICHEVER IS LESS.

FIGURE 5-12 NORMAL SOIL FORCE SECONDARY SOIL MECHANICS METHOD

During the integration of the footpad's equations of motion, the normal soil force is compared to the force required to bring the footpad to rest during the next integration time step. This critical soil force is determined by

$$F_{cr} = m_i \left(g \cos \zeta - \frac{V_x}{\Delta t} \right) - S_x \quad (5-32)$$

where

F_{cr} = critical soil force.

m_i = mass of footpad.

g = local acceleration of gravity.

ζ = ground slope.

V_x = velocity of footpad into landing surface.

Δt = integration step size.

S_x = component of landing gear strut loads normal to landing surface.

The normal soil force applied to the footpad is never allowed to be greater than the magnitude of the force determine by Equation (5-32).

A force, acting in the plane of the landing surface, is obtained by multiplying the normal force by the selected value for the coefficient of friction. This force is applied in a direction opposite to the footpad's velocity in the plane of the landing surface. The components of this inplane soil force are expressed as

$$F_y = - \frac{V_y}{\sqrt{V_y^2 + V_z^2}} \mu F_x$$

$$F_z = - \frac{V_z}{\sqrt{V_y^2 + V_z^2}} \mu F_x \quad (5-33)$$

where

V_y and V_z = inplane footpad velocities as shown in Figure 5-12.

F_y and F_z = components of inplane soil force.

μ = coefficient of friction.

F_x = normal soil force.

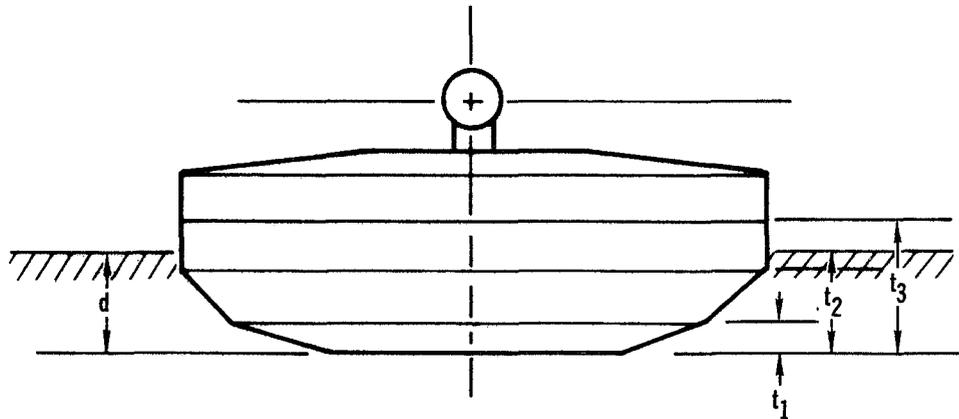
It should be noted that the force, F_x , employed in Equation (5-33), is the magnitude of the normal soil force obtained after the force resulting from the relationships of Figure 5-12 is compared with the critical soil force given by Equation (5-32).

5.1.5 Footpad Attenuation System - An additional attenuation system may be located on the bottom of the footpads to limit the landing loads of the footpads. Provisions have been made for including a crushable material on any or all of the conical segments used to represent the footpad shape. The amount of attenuation material crushing for each footpad is determined by a balance between the attenuator force and the soil force. The footpad attenuation system may be included with either the Primary or Secondary Soil Mechanics Methods. Forces associated with the footpad attenuation system are determined in the soil mechanics subroutine.

The footpad attenuation system may have three crush levels and the pressure-stroke relationship for the attenuator material is shown in Figure 5-13. It is assumed that the attenuator is crushed in a direction normal to the plane of the landing surface. The depth of soil penetration and amount of attenuator crushing, are determined by comparing the crush pressure with the soil pressure. With a soft soil and a stiff footpad attenuation material, most of the deformation will take place in the soil. For a very hard soil, a majority of the deformation will occur in the attenuator material. Intermediate values result in deformation of both the soil and attenuator materials.

At the end of a time interval, the attenuator thickness of each footpad which experiences crushing is adjusted to reflect this deformation. This is done by subtracting the crush distance from the coordinate locating the bottom of the footpad. Thus, if the lander rebounds, the crushed shape of the footpad attenuator is retained for the next impact. When the attenuator on a footpad is completely crushed the footpad attenuation portion of the analysis is bypassed, and the soil forces are applied directly to the footpad.

5.1.6 Footpad Sliding Motion - The two forces determined by the soil mechanics in the plane of the landing surface (F_y and F_z) are components of the inplane soil force F_f which is in a direction opposite to the sliding velocity of the footpad. This force tends to retard the sliding motion of the footpad along the landing surface.



NOTE: THE POINTS DEFINING CHANGES IN ATTENUATOR CRUSH PRESSURE (t_1 , t_2 , & t_3) DO NOT HAVE TO CONFORM TO CHANGES IN THE FOOTPAD GEOMETRY.

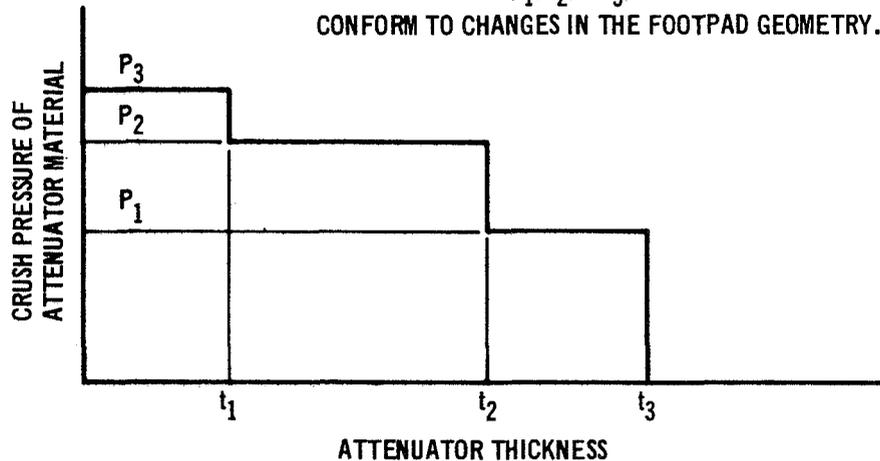


FIGURE 5-13 FOOTPAD ATTENUATION SYSTEM

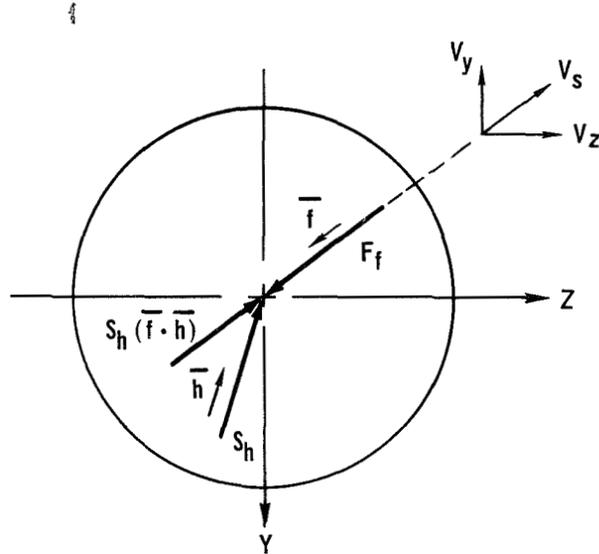
Referring to Figure 5-14, the equation of motion for the footpad in the plane of the landing surface and in the direction of the sliding motion is

$$m_i \frac{dV_s}{dt} = - F_f - S_h (\bar{f} \cdot \bar{h}) \quad (5-34)$$

Therefore, the velocity of the footpad at the end of an integration step is

$$V_s \Big|_{t_n + \Delta t} = - \frac{F_f + S_h (\bar{f} \cdot \bar{h})}{m_i} \Delta t + V_s \Big|_{t_n} \quad (5-35)$$

The critical value of F_f , which will just bring the footpad to rest is obtained from Equation (5-35) by setting the velocity at the end of the integration step ($V_s \Big|_{t_n + \Delta t}$) equal to zero. Thus,



- V_s = SLIDING VELOCITY OF FOOTPAD
 S_h = COMPONENT OF LANDING GEAR STRUT LOAD PARALLEL TO LANDING SURFACE
 \bar{h} = UNIT VECTOR IN DIRECTION OF S_h
 F_f = COMPONENT OF SOIL FORCE IN PLANE OF LANDING SURFACE
 \bar{f} = UNIT VECTOR IN DIRECTION OF F_f
 m_i = MASS OF FOOTPAD
 Δt = INTEGRATION STEP SIZE

WHEN THE FORCES F_f AND S_h OPPOSE EACH OTHER, THE FOLLOWING HOLDS

$$(\bar{f} \cdot \bar{h}) < 0$$

WHILE

$$(\bar{f} \cdot \bar{h}) > 0$$

WHEN THESE FORCES ACT IN THE SAME DIRECTION.

FIGURE 5-14 EVALUATION OF IN-PLANE SOIL FORCE MAGNITUDE

$$F_{fc} = \frac{m_i V_s}{\Delta t} \Big|_{t_n} - S_h (\bar{f} \cdot \bar{h}) \quad (5-36)$$

Equation (5-36), in conjunction with Equation (5-37), is employed to maintain the magnitude of F_f at a value equal to or less than that which is required to bring the footpad to rest during the next integration step. When the relationship of Equation (5-37) holds, the magnitude of F_f determined by

the soil mechanics routine is not changed.

$$F_f \leq F_{fc} \quad (5-37)$$

If the condition of Equation (5-37) is not met, the magnitude of F_f is recalculated by Equation (5-38) when F_f and $S_h(\bar{f} \cdot \bar{h})$ are in opposite directions and

$$F_f = \frac{m_i V_s}{\Delta t} \Big|_{t_n} - S_h (\bar{f} \cdot \bar{h}) \quad (5-38)$$

Equation (5-39) governs the magnitude of F_f when $S_h(\bar{f} \cdot \bar{h})$ and F_f are in the same direction.

$$F_f = \frac{m_i V_s}{\Delta t} \Big|_{t_n} \quad (5-39)$$

5.1.7 Lander Stability - To determine the stability of a legged lander configuration, the "plane of lander motion," as shown in Figure 5-15, is defined. This plane is defined by the gravity vector \bar{g} , and the translational velocity $\dot{\bar{R}}$, of the lander's center of gravity. The lander is considered unstable when the gravity vector passes outside of the area bounded by the lander footpads. If all the lander footpads are on the same side of the plane of lander motion, the lander is said to be experiencing yaw instability.

The more common case is when two footpads are astride the plane of lander motion, as shown in Figure 5-15. In this case, the vector \bar{L} , extending from the lander center of gravity to the intersection point of a line between these two footpads and the plane of lander motion, is obtained. The stability angle is then defined as

$$S = \text{Cos}^{-1} \left\{ \frac{(\bar{L} \cdot \bar{g})}{|\bar{L}| |\bar{g}|} \right\} \quad (5-40)$$

As long as S is positive, the lander is considered to be stable. When S passes through zero the lander is said to be experiencing pitch instability.

To aid in the evaluation of the lander stability, the pitching velocity is also determined. This quantity is the component of the lander's total angular velocity in the plane of lander motion.

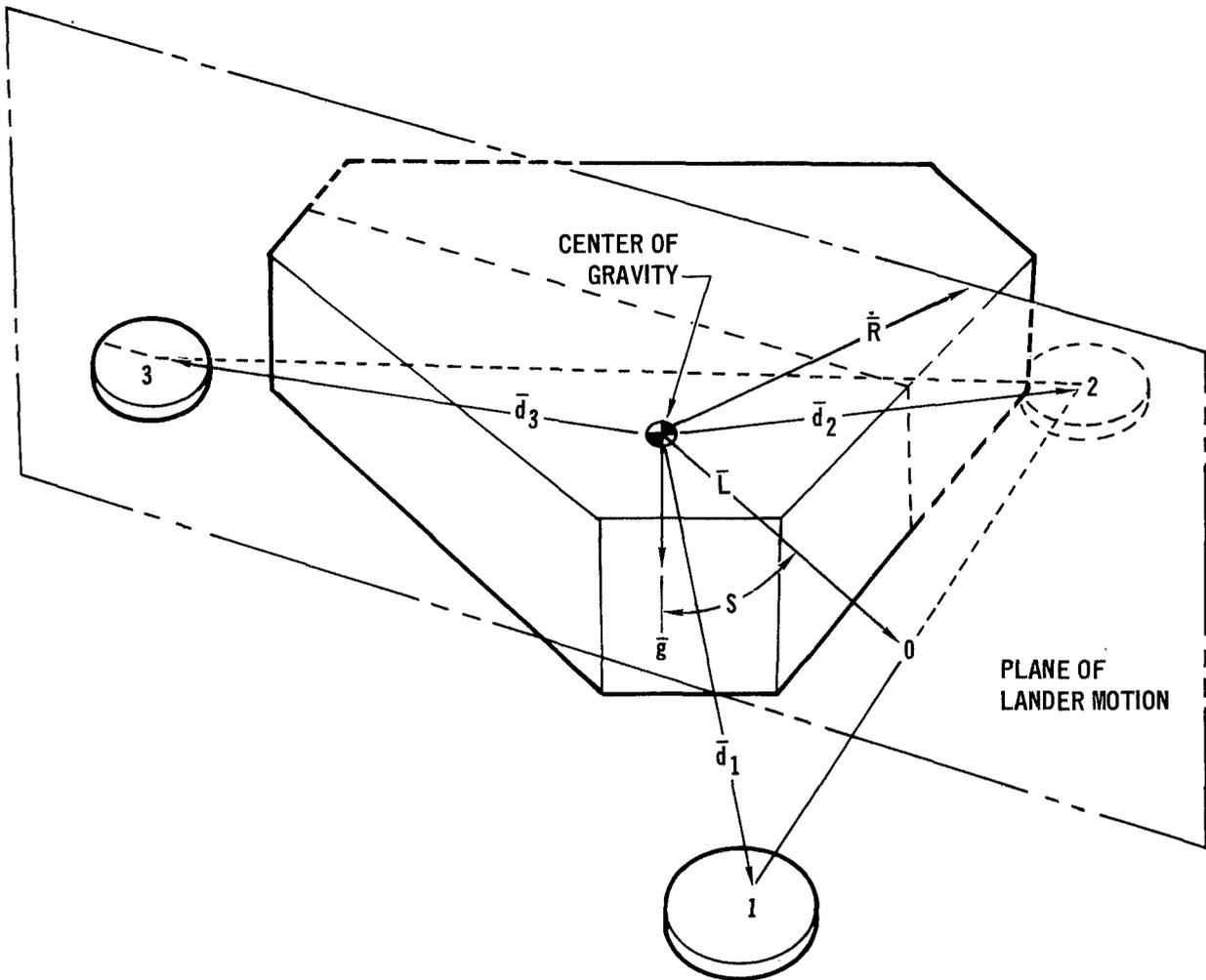


FIGURE 5-15 LANDER STABILITY ANGLE

This stability determination is made in the subroutine STABLE. It is an optional routine of the Landing Loads and Motions Program and is performed if the input indicator JCKSAB is set equal to 1. If the lander becomes unstable, the run is terminated with a printed message stating which type of instability was experienced.

5.2 Program Description - The Landing Loads and Motions Program is best described by defining functions of the program subroutines and examining program organization, as presented in a flow diagram. A listing of the program is given in Appendix I. All programming is in FORTRAN 2.0 for machine computation on CDC 6600 computers.

5.2.1 Subroutines - The Landing Loads and Motions Program is divided into three OVERLAY segments. Each segment consists of an executive subprogram and a number of subroutines as shown in Figure 5-16. This organization has been used to minimize the core storage requirements of the program. Several subroutines have multiple entry points, as indicated in Figure 5-16. The function of each subroutine, depending on the point of entry, is defined in Figure 5-16.

OVERLAY(LLMPT5,0,0) consists of the executive subprogram LLMP. LLMP calls the other two overlays in the proper order and contains all of the COMMON blocks.

READIT is the executive subprogram in OVERLAY(LLMPT5,1,0). This segment of the program reads and prints the input data and initializes all the routines before integration of the equations of motion.

LLMPEX is the executive subprogram in OVERLAY(LLMPT5,2,0). It controls the solution of the equations of motion. Subroutines in this portion of the program obtain the forces in the landing gear struts and determine the soil forces. These forces are summed on the center body and contacting footpads, resulting accelerations determined, equations of motion integrated, and time history quantities printed. At the completion of a time history, program control is returned to OVERLAY(LLMPT5,0,0) for the possible consideration of an additional data set.

Two numerical integration methods are incorporated in the subroutine RKCUT. These consist of a constant step Runge-Kutta method and a variable step Runge-Kutta method. A description of these integration techniques is presented in Appendix F.

SUBPROGRAM NAME	SUBROUTINE NAME	ENTRY POINT	OVERLAY LOCATION	SUBPROGRAM OR SUBROUTINE OPERATIONS
LLMP		LLMP	OVERLAY (LLMPT5,0,0)	MAIN EXECUTIVE SUBPROGRAM. CONTROLS THE EXECUTION OF THE TOTAL PROGRAM.
READIT		READIT	OVERLAY (LLMPT5,1,0)	EXECUTIVE SUBPROGRAM IN OVERLAY (LLMPT5,1,0). ALSO READS THE INPUT DATA AND DETERMINES IF THE AMOUNT OF DATA IS CONSISTENT WITH THE INPUT CONTROL INDICATORS. IF NOT, THE PROGRAM IS TERMINATED. INITIALIZES THE PROGRAM.
	DATAOT	DATAOT		PRINTS INPUT DATA.
LLMPEX		LLMPEX		EXECUTIVE SUBPROGRAM IN OVERLAY (LLMPT5,2,0). CONTROLS INTEGRATION OF EQUATIONS OF MOTION.
	FTPAD	FTPAD		CONTROLS DETERMINATION OF LOADS IN LANDING GEAR STRUTS AND SOIL FORCES.
	GEOM	GEOM		DETERMINES DIRECTION COSINE MATRIX AND TIME DERIVATIVES OF EULER ANGLES.
	STRUT SOIL INITUP	STRUT SOIL		DETERMINES LOAD IN LANDING GEAR STRUTS. DETERMINES SOIL FORCES.
		LOC INUPD		NUMERICAL INTEGRATION INITIALIZATION SUBROUTINE.
	RKCUT		OVERLAY (LLMPT5,2,0)	DEFINES AND STORES CUTOFF VARIABLES. SETS UP STORAGE FOR INTEGRATED VARIABLES. RUNGE-KUTTA NUMERICAL INTEGRATION SUBROUTINE.
		SETUP INTEG UPDATE		INITIALIZATION OF RKCUT ROUTINE. INTEGRATION OF EQUATIONS OF MOTION.
	OUTPUT	CUT OUTPUT SUMMRY		UPDATES INTEGRATION VARIABLES AND MODIFIES INTEGRATION INTERVAL. CHECKS CUTOFF LIMITS. PRINTS TIME HISTORY INFORMATION. PRINTS SUMMARY INFORMATION AT THE END OF A DATA CASE.
	STABLE MATINV GMPRD	STABLE MATINV GMPRD		CHECKS STABILITY OF LANDER. MATRIX INVERSION SUBROUTINE. MATRIX MULTIPLICATION SUBROUTINE.

**FIGURE 5-16 LANDING LOADS AND MOTIONS PROGRAM
SUBROUTINES AND SUBPROGRAMS**

5.2.2 Flow Diagram - A flow diagram showing the general operation of the Landing Loads and Motions Program is presented in Figure 5-17. The three OVERLAY segments are shown in addition to the various subprograms and sub-routines which are located in each OVERLAY. This diagram is not intended to be a comprehensive programming chart, but shows the general flow of the program logic and indicates the order of operations within each subroutine. A complete listing of the Landing Loads and Motions Program is given in Appendix I.

5.3 Program Operation - Successful operation of the Landing Loads and Motions Program depends on proper input of data and correct interpretation of output data. These considerations are discussed in the following paragraphs. Examples of required input data and resulting output data for a typical landing condition are given in Appendix D.

5.3.1 Input Data - Required as input data is information describing the geometric and inertia properties of the specific lander to be studied; initial lander attitudes; linear and rotational velocities; surface conditions such as ground slope and soil properties; and the indicators needed to control the program's operation. This section defines the format of the input data cards and contains instructions for properly supplying input data to the program.

Figure 5-18 shows the required format for the input data cards. Columns 6 through 9 contain a card number, which must be right justified. Input data is placed in floating point form in columns 11-20, 21-30, 31-40, 41-50, and 51-60. Data input in a F format need not be right justified, but data in an E format must be right justified. In either case, the data must be contained entirely within the field of 10 columns provided. Columns 61 and on may be used for sequence numbers, identification statements, or comments. Following the last card of a data set, a card with NEXT in columns 1 through 4, must appear. Multiple cases may be run by stacking the data sets. Note that the input indicator IDSETN(Card 8) must be equal to 1 if another data set follows the current data set. The data cards for any additional data sets follow the first data set, and each of these are terminated with a NEXT in columns 1 through 4. A card with STOP in columns 1 through 4 signals the end of all the data sets.

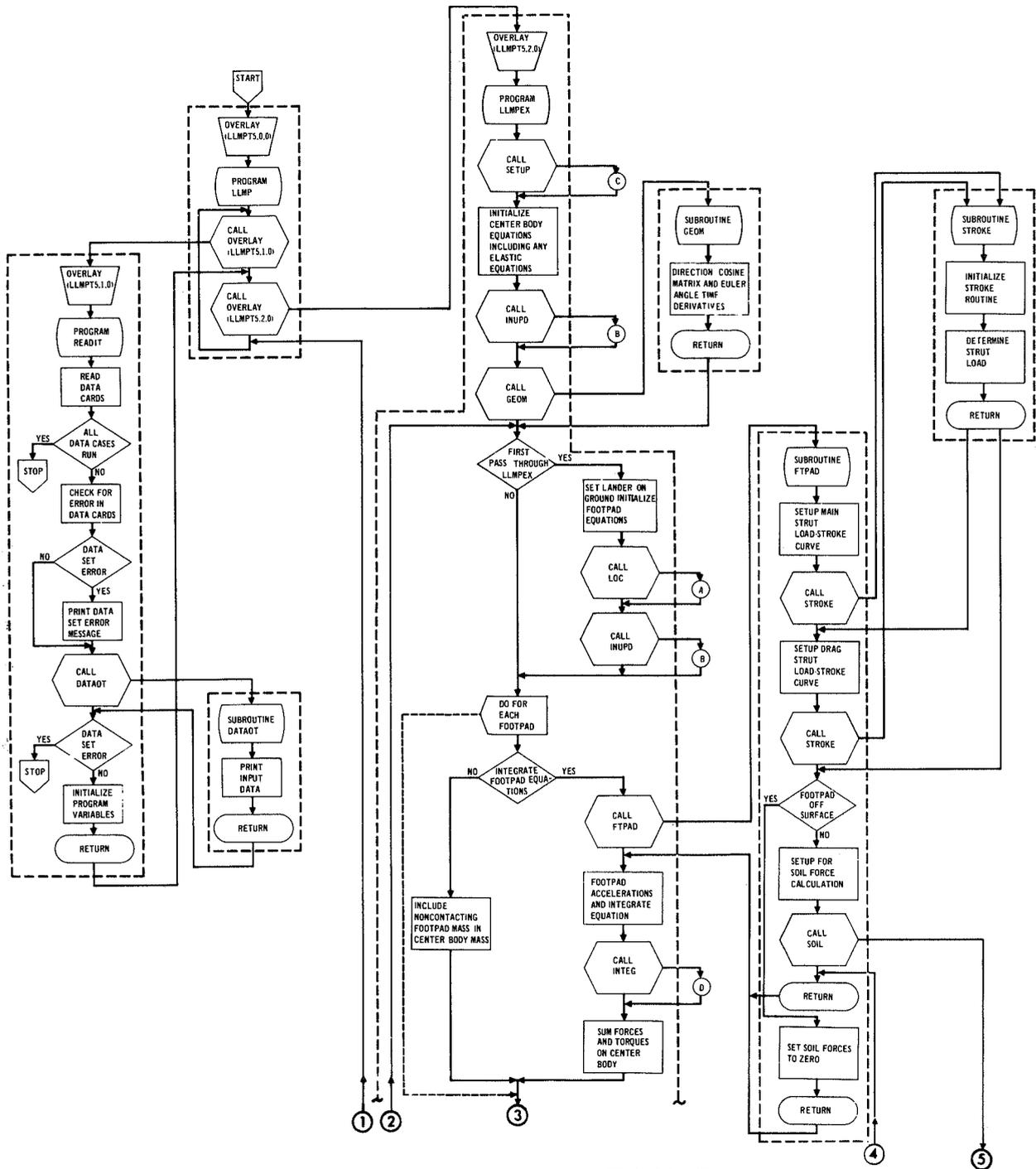


FIGURE 5-17 FLOW DIAGRAM LANDING LOADS AND MOTIONS PROGRAM

(CONTINUED FROM PREVIOUS PAGE)

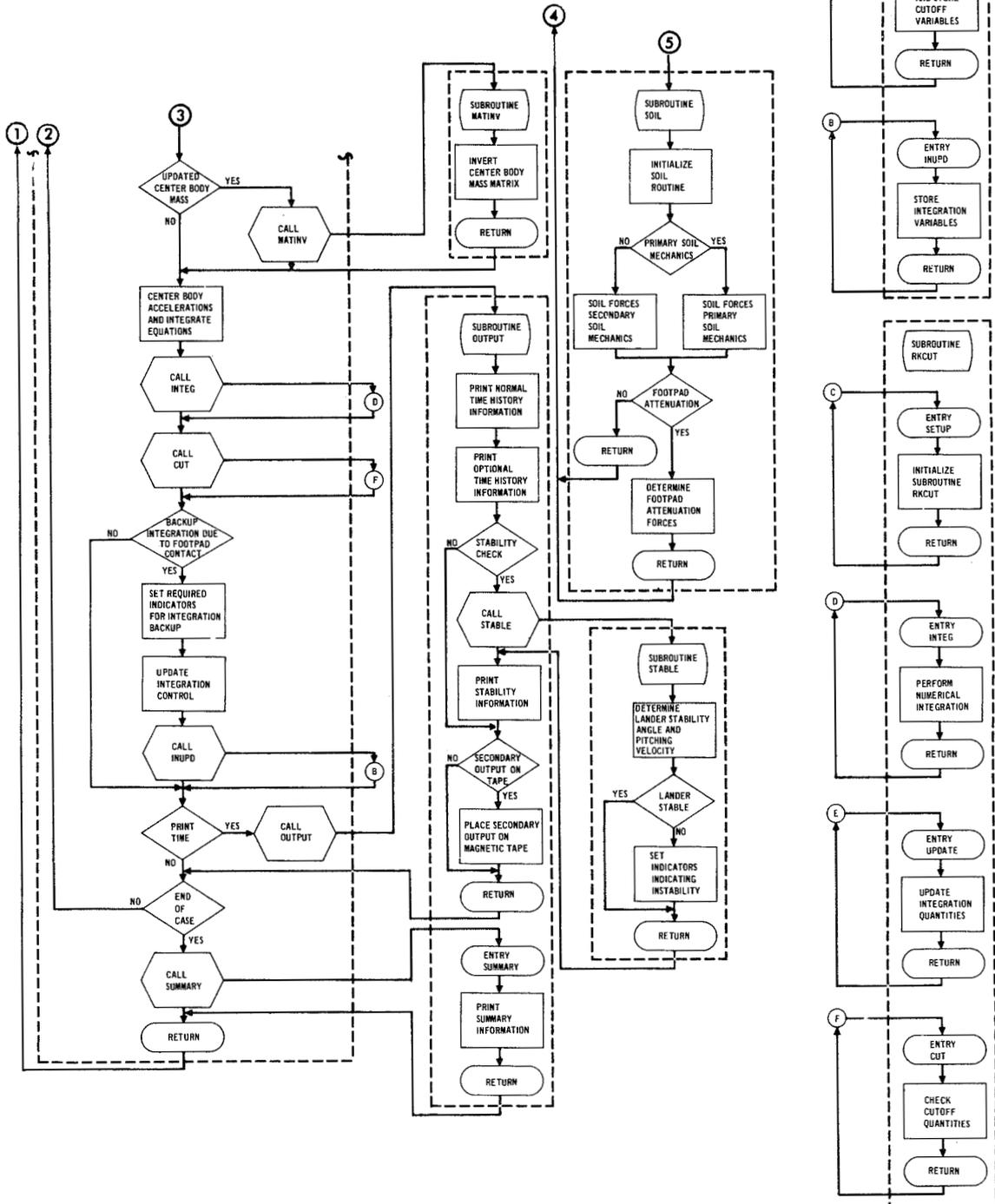


FIGURE 5-17 FLOW DIAGRAM LANDING LOADS AND MOTIONS PROGRAM (Continued)

There are a number of indicators and counters which check the input data as it is read to ensure that the correct amount of information has been input. If the number of data cards is incorrect, the run will be terminated, and error messages printed to indicate where the data error occurred. All of the indicators governing these input options are discussed in Figure 5-19.

Data cards for the first data set must contain all of the information required to completely initialize the first case. Only optional data, consistent with the input control indicators, may be left out of the first data set. For following data sets, only the information which is to be different (or in addition to that of the preceding case) needs to be changed on the appropriate cards.

There is no specific order in which the low number data cards, Cards 1 through 42, must appear in a data set. However, the order of the higher numbered cards (Cards 100 through 1700) is of importance. All of these cards with higher numbers define variables which are subscripted in the Landing Loads and Motions Program. Therefore, all the cards with the same card number must be input in the order in which the user requires this information stored in the program.

All input parameters and their associated data card numbers are defined in Figure 5-20. Most of these parameters are adequately explained in this figure, and Figure 5-19, but a number require additional comments.

There is no specific system of units associated with the input information, except for the angular quantities which must be expressed in degrees. All other parameters may be expressed in any consistent set of units, either English or Metric (inches or centimeters, pounds or dynes).

Care must be exercised to ensure that lander velocity, position, and attitude are correctly initialized. All lander initial conditions are referenced to the Gravity Coordinate System, Figure 5-1. The three components of velocity VELX, VELY, and VELZ are parallel to the axes of the Gravity Coordinate System. Note that a negative VELX is directed into the landing surface.

The desired ground slope, relating the Surface Coordinate System to the Gravity Coordinate System, must be established. This is accomplished by rotating the Surface Coordinate System about the Gravity Coordinate System Y_g axis an amount ζ equal to the ground slope. A positive slope corresponds to a rotation in the positive sense about Y_g .

The lander's initial angular orientation is obtained by three successive rotations about the three lander axes. The first is a rotation, ANGX, of the Lander Coordinate System about the Gravity Coordinate System X_g axis. This is followed by a rotation, ANGY, about the displaced position of the Lander Coordinate System Y axis. Final angular orientation is obtained by the rotation, ANGZ, about the direction of the Z axis resulting from the first two rotations. When these three rotations are zero, the initial attitude of the Lander Coordinate System is aligned with the Gravity Coordinate System.

The input quantity CUTERR (Card 7) defines a tolerance band, above the landing surface, which governs the placement of a footpad's equations of motion into the integration routine. When a footpad enters this band, the inertia effects of the footpad are removed from the center body equations of motion, initial conditions for the footpad are determined by center body motions, and integration of footpad equations of motion begun. This tolerance parameter has meaning only when INLEG (Card 2) is 0.

When including the effects of an elastic center body structure, the indicator MODEIN (Card 42) governs the number of center body modes which are input. NMODES (Card 3) governs the number of modes actually included in the analysis, and may be less than or equal to MODEIN.

A discussion of the input data required for a typical landing condition is presented in Sections 5.3.3 and 6.2.2 and the computer output for this case is given in Appendix D.

5.3.2 Output Data - At specified times during the integration of the equations of motion, various time varying quantities defining the position, velocities, and accelerations of the lander center body and footpads are printed. At the completion of a run, a summary page listing the maximum strokes in the landing gear struts, the reason for termination of the run, and

CARD 2:	IPTCNT	INDICATOR WHICH DEFINES NUMBER OF INTEGRATION STEPS BETWEEN PRINT TIMES.
	INLEG = 0	FOOTPAD EQUATIONS OF MOTION INTEGRATED ONLY AFTER ACTUAL IMPACT WITH LANDING SURFACE.
	INLEG = 1	FOOTPAD EQUATIONS OF MOTION INTEGRATED AT ALL TIMES.
	IFPRT > 0	NUMBER OF INTEGRATION TIME STEPS TO BE PRINTED FOLLOWING IMPACT OF A FOOTPAD.
	IFPRT = 0	IPTCNT (CARD 2) CONTROLS PRINT INTERVAL AT ALL TIMES.
CARD 3:	NMODES = 0	RIGID CENTER BODY ASSUMED.
	NMODES > 0	NUMBER OF CENTER BODY MODES INCLUDED IN ANALYSIS (5 MAXIMUM).
	NOOUT = 0	NO SECONDARY ACCELERATION OUTPUT POINTS INCLUDED.
	NOOUT > 0	NUMBER OF SECONDARY ACCELERATION OUTPUT POINTS (10 MAXIMUM).
CARD 8:	NFORC = 0	NO SECONDARY TIME HISTORY OUTPUT OBTAINED.
	NFORC = 1	SECONDARY TIME HISTORY PLACED ON MAGNETIC TAPE CORRESPONDING TO TIMES OF PRINTED OUTPUT.
	IQUOUT = 0	NO SECONDARY INTEGRATION VARIABLE OUTPUT.
	IQUOUT = 1	SECONDARY INTEGRATION VARIABLE OUTPUT OBTAINED WITH NORMAL OUTPUT. THIS OPTION IS OVERRIDDEN WHEN NMODES = 0 SINCE THE NORMAL OUTPUT LISTS ALL THE INTEGRATED VARIABLES IN THIS CASE.
	JKSAB = 0	NO LANDER STABILITY CHECK.
	JKSAB = 1	LANDER STABILITY CHECK PERFORMED.
	IDSETN = 0	INDICATES ANOTHER DATA CASE DOES NOT FOLLOW THE CURRENT DATA.
	IDSETN = 1	INDICATES ANOTHER DATA CASE DOES FOLLOW THE CURRENT DATA.
CARD 21:	NTYPE = 0	PRIMARY SOIL MECHANICS ROUTINE.
	NTYPE = 1	SECONDARY SOIL MECHANICS ROUTINE.
CARD 23:	NOLEG	INDICATES NUMBER OF LEGS (5 MAXIMUM).
	ILEG = 0	INVERTED TRIPOD GEAR.
	ILEG = 1	CANTILEVER GEAR.
CARD 42:	MODEIN = 0	NO MODAL DATA INPUT.
	MODEIN > 0	NUMBER OF CENTER BODY MODES TO BE INPUT (5 MAXIMUM).
CARD 100:		NUMBER OF INPUT CARDS 100 MUST EQUAL NOOUT (CARD 3).
CARD 200:		NUMBER OF INPUT CARDS 200 MUST EQUAL NOLEG (CARD 23).
CARD 300:		NUMBER OF INPUT CARDS 300 MUST EQUAL NOLEG (CARD 23).
CARD 400:		NUMBER OF INPUT CARDS 400 MUST EQUAL TWO TIMES NOLEG (CARD 23).
CARD 500:		NUMBER OF INPUT CARDS 500 MUST EQUAL MODEIN (CARD 42).
CARD 600:		NUMBER OF INPUT CARDS 600 MUST EQUAL MODEIN (CARD 42).
CARD 700:		NUMBER OF INPUT CARDS 700 MUST EQUAL MODEIN (CARD 42).
CARD 800:		NUMBER OF INPUT CARDS 800 MUST EQUAL NOLEG (CARD 23).
CARD 900:		NUMBER OF INPUT CARDS 900 MUST EQUAL NOLEG (CARD 23).
CARD 1000:		NUMBER OF INPUT CARDS 1000 MUST EQUAL NOLEG (CARD 23).
CARD 1100:		NUMBER OF INPUT CARDS 1100 MUST EQUAL TWO TIMES NOLEG (CARD 23).
CARD 1200:		NUMBER OF INPUT CARDS 1200 MUST EQUAL TWO TIMES NOLEG (CARD 23).
CARD 1300:		NUMBER OF INPUT CARDS 1300 MUST EQUAL TWO TIMES NOLEG (CARD 23).
CARD 1400:		NUMBER OF INPUT CARDS 1400 MUST EQUAL NOOUT (CARD 3).
CARD 1500:		NUMBER OF INPUT CARDS 1500 MUST EQUAL NOOUT (CARD 3).
CARD 1600:		NUMBER OF INPUT CARDS 1600 MUST EQUAL NOOUT (CARD 3).
CARD 1700:		NUMBER OF INPUT CARDS 1700 MUST EQUAL MODEIN (CARD 42).

FIGURE 5-19 INPUT DATA CONTROL INDICATORS LANDING LOADS AND MOTIONS PROGRAM

CARD NO.	INPUT VARIABLE	COORDINATE SYSTEM*	VARIABLE DEFINITION
1	CASENO		CASE NUMBER.
2	TIMAX		MAXIMUM RUN TIME.
2	IPTCNT		TIME HISTORY PRINT CONTROL. NUMBER OF Δt 's BETWEEN PRINTS.
2	INLEG		FOOTPAD INTEGRATION CONTROL INDICATOR. = 0 FOOTPAD EQUATIONS INTEGRATED ONLY AFTER FOOTPAD IMPACT = 1 FOOTPAD EQUATIONS INTEGRATED AT ALL TIMES
2	IFPRT		PRINT CONTROL INDICATOR FOR FOOTPAD IMPACT = 0 CONTROL OF PRINT INTERVAL ALWAYS GOVERNED BY IPTCNT > 0 NUMBER INTEGRATION TIME STEPS TO BE PRINTED FOLLOWING FOOTPAD IMPACT
3	NMODES		NUMBER OF ELASTIC MODES INCLUDED IN ANALYSIS.
3	NOOUT		NUMBER OF SECONDARY ACCELERATION OUTPUT POINTS.
4	INDFXD		INDICATOR TO SUPPRESS CENTER BODY X_S DEGREE OF FREEDOM. = 0 ALLOW DEGREE OF FREEDOM $\neq 0$ SUPPRESS DEGREE OF FREEDOM
4	INDFYD		INDICATOR TO SUPPRESS CENTER BODY Y_S DEGREE OF FREEDOM (SEE INDFXD).
4	INDFZD		INDICATOR TO SUPPRESS CENTER BODY Z_S DEGREE OF FREEDOM (SEE INDFXD).
5	INDFXR		INDICATOR TO SUPPRESS CENTER BODY ROTATION ABOUT X AXIS (SEE INDFXD).
5	INDFYR		INDICATOR TO SUPPRESS CENTER BODY ROTATION ABOUT Y AXIS (SEE INDFXD).
5	INDFZR		INDICATOR TO SUPPRESS CENTER BODY ROTATION ABOUT Z AXIS (SEE INDFXD).
6	HMAX		MAXIMUM INTEGRATION TIME INTERVAL.
6	HMIN		MINIMUM INTEGRATION TIME INTERVAL (VARIABLE STEP RUNGE-KUTTA).
6	EMAX		MAXIMUM INTEGRATION ACCURACY (VARIABLE STEP RUNGE-KUTTA).
6	EMIN		MINIMUM INTEGRATION ACCURACY (VARIABLE STEP RUNGE-KUTTA).
6	IP		QUANTITY USED TO SET INITIAL INTEGRATION STEP SIZE. $\Delta t = \Delta t_{max} / (2^{IP})$.
7	IVARH		INTEGRATION INDICATOR IVARH = 0 VARIABLE STEP INTEGRATION, IVARH = 1 CONSTANT STEP INTEGRATION.
7	IMTH		DUMMY VARIABLE. SET EQUAL TO 0.
7	CUTERR		TOLERANCE ON CONTROL OF FOOTPAD IMPACT
8	NFORC		SECONDARY TIME HISTORY OUTPUT INDICATOR. NFORC = 1 SECONDARY OUTPUT ON TAPE 3. NFORC = 0 NO SECONDARY OUTPUT.
8	IQUOUT		INTEGRATED VARIABLE OUTPUT INDICATOR. IQUOUT = 1 PRINT ALL INTEGRATED VARIABLES. IQUOUT = 0 NO INTEGRATED VARIABLE OUTPUT.
8	JCKSAB		STABILITY CHECK INDICATOR. JCKSAB = 0 - DO NOT CHECK STABILITY. JCKSAB = 1 - CHECK LANDER STABILITY.
8	IDSETN		MULTIPLE DATA CASE INDICATOR. IDSETN = 1 - ANOTHER DATA SET FOLLOWING CURRENT DATA SET. IDSETN = 0 - NO ADDITIONAL DATA SETS.

* NOTE: THE FOLLOWING ABBREVIATIONS ARE USED TO DEFINE THE COORDINATE SYSTEMS.
SCS - SURFACE COORDINATE SYSTEM.
LCS - LANDER COORDINATE SYSTEM.
GCS - GRAVITY COORDINATE SYSTEM.

FIGURE 5-20 INPUT DATA LANDING LOADS AND MOTIONS PROGRAM

CARD NO.	INPUT VARIABLE	COORDINATE SYSTEM*	VARIABLE DEFINITION
9	ZETA	GCS	GROUND SLOPE.
9	GRAV		ACCELERATION OF GRAVITY ON PLANET.
9	GRAVE		ACCELERATION OF GRAVITY ON EARTH.
10	ANGX	LCS	INITIAL ANGULAR ROTATION OF LCS ABOUT X_g .
10	ANGY	LCS	ANGULAR ROTATION OF LCS ABOUT DIRECTION OF Y FOLLOWING ANGX.
10	ANGZ	LCS	ANGULAR ROTATION OF LCS ABOUT DIRECTION OF Z FOLLOWING ANGX AND ANGY.
11	WX	LCS	INITIAL LANDER ANGULAR VELOCITY ABOUT X AXIS.
11	WY	LCS	INITIAL LANDER ANGULAR VELOCITY ABOUT Y AXIS.
11	WZ	LCS	INITIAL LANDER ANGULAR VELOCITY ABOUT Z AXIS.
12	VELX	GCS	INITIAL LANDER VELOCITY ALONG X_g AXIS.
12	VELY	GCS	INITIAL LANDER VELOCITY ALONG Y_g AXIS.
12	VELZ	GCS	INITIAL LANDER VELOCITY ALONG Z_g AXIS.
13	CBMASS		CENTER BODY MASS.
14	CBIXX		CENTER BODY MASS MOMENT OF INERTIA - I_{xx} .
14	CBIIY		CENTER BODY MASS MOMENT OF INERTIA - I_{yy} .
14	CBIZZ		CENTER BODY MASS MOMENT OF INERTIA - I_{zz} .
15	CBIXY		CENTER BODY PRODUCT OF INERTIA - I_{xy} .
15	CBIXZ		CENTER BODY PRODUCT OF INERTIA - I_{xz} .
15	CBIZY		CENTER BODY PRODUCT OF INERTIA - I_{yz} .
16	FPMASS		FOOTPAD MASS.
17	RAD(I)		RADIUS OF ITH FOOTPAD SEGMENT (SEE FIGURE 5-8).
18	SS(I)		DISTANCE FROM ITH FOOTPAD SEGMENT TO FOOTPAD PIVOT POINT (SEE FIGURE 5-8).
19	ATTHCK(I)		THICKNESS OF ITH SEGMENT OF FOOTPAD ATTENUATION MATERIAL RELATIVE TO BOTTOM OF FOOTPAD (SEE FIGURE 5-13).
20	ATTPRS(I)		CRUSH PRESSURE OF ITH SEGMENT OF FOOTPAD ATTENUATION MATERIAL (SEE FIGURE 5-13).
21	NTYPE		SOIL MECHANICS INDICATOR. NTYPE = 0 - PRIMARY SOIL MECHANICS. NTYPE = 1 - SECONDARY SOIL MECHANICS.
22	SOILP(1)		FOR NTYPE = 0 - SOIL INTERNAL FRICTION ANGLE. FOR NTYPE = 1 - COEFFICIENT OF FRICTION.
22	SOILP(2)		FOR NTYPE = 0 - SOIL UNIT WEIGHT. FOR NTYPE = 1 - SLOPE OF ELASTIC SOIL PENETRATION RELATIONSHIP.
22	SOILP(3)		FOR NTYPE = 0 - SOIL RELATIVE DENSITY. FOR NTYPE = 1 - MAXIMUM SOIL PRESSURE.
23	NOLEG		NUMBER OF LEGS.
23	ILEG		TYPE GEAR INDICATOR. ILEG = 0 - INVERTED TRIPOD. ILEG = 1 - CANTILEVER.
23	DRAGST		DISTANCE FROM CENTER BODY ATTACH POINT TO DRAG STRUT ATTACH POINT ALONG AXIS OF MAIN STRUT FOR CANTILEVER GEAR.
24	PFCMS(I)		MAIN STRUT ITH COMPRESSION PLASTIC LOAD LEVEL (SEE FIGURE 4-18)
25	PFTMS(I)		MAIN STRUT ITH TENSION PLASTIC LOAD LEVEL (SEE FIGURE 4-18).

*NOTE: THE FOLLOWING ABBREVIATIONS ARE USED TO DEFINE THE COORDINATE SYSTEMS.

SCS - SURFACE COORDINATE SYSTEM.

LCS - LANDER COORDINATE SYSTEM.

GCS - GRAVITY COORDINATE SYSTEM.

FIGURE 5-20 INPUT DATA LANDING LOADS AND MOTIONS PROGRAM (Continued)

CARD NO.	INPUT VARIABLE	COORDINATE SYSTEM*	VARIABLE DEFINITION
26	CDCMS(I)		MAIN STRUT ITH COMPRESSION PLASTIC STROKE LIMIT (SEE FIGURE 4-18).
27	CDTMS(I)		MAIN STRUT ITH TENSION PLASTIC STROKE LIMIT (SEE FIGURE 4-18).
28	SRCMS(I)		MAIN STRUT ITH COMPRESSION SPRING RATE (SEE FIGURE 4-18).
29	SRTMS(I)		MAIN STRUT ITH TENSION SPRING RATE (SEE FIGURE 4-18).
30	SCMXMS		MAXIMUM ALLOWABLE MAIN STRUT COMPRESSION STROKE.
30	STMXMS		MAXIMUM ALLOWABLE MAIN STRUT TENSION STROKE.
30	SRUCMS		MAIN STRUT COMPRESSION UNLOADING SPRING RATE.
30	SRUTMS		MAIN STRUT TENSION UNLOADING SPRING RATE.
31	IRETMS		MAIN STRUT UNLOADING INDICATOR. IRETMS = 0 - UNLOAD ALONG SRUCMS OR SRUTMS. IRETMS = 1 - UNLOADING SPRING RATE OBTAINED FROM LOAD-STROKE CURVE.
32	FRICMS		MAIN STRUT SLIDING FRICTION FORCE.
32	COEFMS		COEFFICIENT OF MAIN STRUT VELOCITY DEPENDENT FRICTION.
32	GAMMS		POWER OF MAIN STRUT VELOCITY DEPENDENT FRICTION.
32	AEI1		BENDING STIFFNESS (EI) OF UPPER SECTION OF CANTILEVER GEAR MAIN STRUT. (SEE FIGURE 5-5)
32	AEI2		BENDING STIFFNESS (EI) OF LOWER SECTION OF CANTILEVER GEAR MAIN STRUT (SEE FIGURE 5-5).
33	PFCDS(I)		DRAG STRUT ITH COMPRESSION PLASTIC LOAD LEVEL (SEE FIG. 4-18)
34	PFTDS(I)		DRAG STRUT ITH TENSION PLASTIC LOAD LEVEL (SEE FIG. 4-18)
35	CDCDS(I)		DRAG STRUT ITH COMPRESSION PLASTIC STROKE LIMIT (SEE FIGURE 4-18).
36	CDTDS(I)		DRAG STRUT ITH TENSION PLASTIC STROKE LIMIT (SEE FIGURE 4-18).
37	SRCDS(I)		DRAG STRUT ITH COMPRESSION SPRING RATE (SEE FIGURE 4-18).
38	SRTDS(I)		DRAG STRUT ITH TENSION SPRING RATE (SEE FIGURE 4-18)
39	SCMXDS		MAXIMUM ALLOWABLE DRAG STRUT COMPRESSION STROKE.
39	STMXDS		MAXIMUM ALLOWABLE DRAG STRUT TENSION STROKE.
39	SRUCDS		DRAG STRUT COMPRESSION UNLOADING SPRING RATE.
39	SRUTDS		DRAG STRUT TENSION UNLOADING SPRING RATE.
40	IRETDS		DRAG STRUT UNLOADING INDICATOR. IRETDS = 0 - UNLOAD ALONG SRUCDS OR SRUTDS. IRETDS = 1 - UNLOADING SPRING RATE OBTAINED FROM LOAD-STROKE CURVE.
41	FRICDS		DRAG STRUT SLIDING FRICTION FORCE.
41	COEFDS		COEFFICIENT OF DRAG STRUT VELOCITY DEPENDENT FRICTION.
41	GAMDS		POWER OF DRAG STRUT VELOCITY DEPENDENT FRICTION.
42	MODEIN		NUMBER OF ELASTIC MODES INPUT.
100	XOUT(I)	LCS	X COORDINATE OF ITH ACCELERATION OUTPUT POINT.
100	YOUT(I)	LCS	Y COORDINATE OF ITH ACCELERATION OUTPUT POINT.
100	ZOUT(I)	LCS	Z COORDINATE OF ITH ACCELERATION OUTPUT POINT.

*NOTE: THE FOLLOWING ABBREVIATIONS ARE USED TO DEFINE THE COORDINATE SYSTEMS.
SCS - SURFACE COORDINATE SYSTEM.
LCS - LANDER COORDINATE SYSTEM.
GCS - GRAVITY COORDINATE SYSTEM

FIGURE 5-20 INPUT DATA LANDING LOADS AND MOTIONS PROGRAM (Continued)

CARD NO.	INPUT VARIABLE	COORDINATE SYSTEM*	VARIABLE DEFINITION
200	XFP(I)	LCS	X COORDINATE OF I TH FOOTPAD.
200	YFP(I)	LCS	Y COORDINATE OF I TH FOOTPAD.
200	ZFP(I)	LCS	Z COORDINATE OF I TH FOOTPAD.
300	XMSCB(I)	LCS	X COORDINATE OF CENTER BODY END OF I TH MAIN STRUT.
300	YMSCB(I)	LCS	Y COORDINATE OF CENTER BODY END OF I TH MAIN STRUT.
300	ZMSCB(I)	LCS	Z COORDINATE OF CENTER BODY END OF I TH MAIN STRUT.
400	XDSCB(I)	LCS	X COORDINATE OF CENTER BODY END OF I TH DRAG STRUT.
400	YDSCB(I)	LCS	Y COORDINATE OF CENTER BODY END OF I TH DRAG STRUT.
400	ZDSCB(I)	LCS	Z COORDINATE OF CENTER BODY END OF I TH DRAG STRUT.
500	GM(I)		GENERALIZED MASS OF I TH ELASTIC MODE. (EQUATION (5-13))
500	OMEGA(I)		FREQUENCY OF I TH ELASTIC MODE (Hz).
600	WNX(I)		GENERALIZED INERTIA PROPERTY N_{xn} FOR I TH ELASTIC MODE (EQUATION (5-13)).
600	WNY(I)		GENERALIZED INERTIA PROPERTY N_{yn} FOR I TH ELASTIC MODE (EQUATION (5-13)).
600	WNZ(I)		GENERALIZED INERTIA PROPERTY N_{zn} FOR I TH ELASTIC MODE (EQUATION (5-13)).
700	PX(I)		GENERALIZED INERTIA PROPERTY P_{xn} FOR I TH ELASTIC MODE (EQUATION (5-13)).
700	PY(I)		GENERALIZED INERTIA PROPERTY P_{yn} FOR I TH ELASTIC MODE (EQUATION (5-13)).
700	PZ(I)		GENERALIZED INERTIA PROPERTY P_{zn} FOR I TH ELASTIC MODE (EQUATION (5-13)).
800	PMSX(I,J)	LCS	X MODE SHAPE FOR I TH MAIN STRUT POINT IN J TH MODE.
900	PMSY(I,J)	LCS	Y MODE SHAPE FOR I TH MAIN STRUT POINT IN J TH MODE.
1000	PMSZ(I,J)	LCS	Z MODE SHAPE FOR I TH MAIN STRUT POINT IN J TH MODE.
1100	PDSX(I,J)	LCS	X MODE SHAPE FOR I TH DRAG STRUT POINT IN J TH MODE.
1200	PDSY(I,J)	LCS	Y MODE SHAPE FOR I TH DRAG STRUT POINT IN J TH MODE.
1300	PDSZ(I,J)	LCS	Z MODE SHAPE FOR I TH DRAG STRUT POINT IN J TH MODE.
1400	POUTX(I,J)	LCS	X MODE SHAPE FOR I TH ACCELERATION POINT IN J TH MODE.
1500	POUTY(I,J)	LCS	Y MODE SHAPE FOR I TH ACCELERATION POINT IN J TH MODE.
1600	POUTZ(I,J)	LCS	Z MODE SHAPE FOR I TH ACCELERATION POINT IN J TH MODE.
1700	PCGX(I)	LCS	X MODE SHAPE AT CENTER OF GRAVITY IN I TH MODE.
1700	PCGY(I)	LCS	Y MODE SHAPE AT CENTER OF GRAVITY IN I TH MODE.
1700	PCGZ(I)	LCS	Z MODE SHAPE AT CENTER OF GRAVITY IN I TH MODE.

*NOTE: THE FOLLOWING ABBREVIATIONS ARE USED TO DEFINE THE COORDINATE SYSTEMS.
SCS - SURFACE COORDINATE SYSTEM.
LCS - LANDER COORDINATE SYSTEM.
GCS - GRAVITY COORDINATE SYSTEM.

FIGURE 5-20 INPUT DATA LANDING LOADS AND MOTIONS PROGRAM (Continued)

case run time are presented. In addition, optional output data, as requested through input indicators, may be obtained. These consist of lander stability angle and pitching velocity, all of the individual integrated quantities when considering a flexible center body, and accelerations at points other than the center body center of gravity. In this latter case, the coordinates of the points at which the accelerations are desired are included as input data. The individual integrated quantities are of interest when considering a flexible structure because the normal output presents the combination of the rigid body and elastic motions. Examples of the output data obtained from the Landing Loads and Motions Program are presented in Appendix D.

The units of the output data are consistent with the system of units used in the input data. Only the angular quantities are output in the specific units of degrees.

An additional output option is available in the Landing Loads and Motions Program. This option allows the output of the landing gear strut forces and all of the time history variables required to determine the inertia loading throughout the center body structure. With this information, the internal loads of the center body structural members may be determined with the Center Body Option of the Structural Analysis Program. This data is output at points in time corresponding to the times when printed output is generated by the program. This option results when the input indicator NFORC (Card 8) is equal to 1 and the data is placed on magnetic tape for future use. The Center Body Landing Loads Program, which is used to retrieve this data from the tape and generate input information for the Structural Analysis Program, is discussed in Appendix E.

5.3.3 Example of Program Operation - A listing of the input data, and the resulting printed output obtained from the Landing Loads and Motions Program is presented in Appendix D. The output pages in this appendix present the input data, the beginning of the time history, and the last pages of the time history including the summary information. This data set corresponds to a computer run demonstrating correlation with the third drop test conducted at NASA Langley Research Center on the Task Order Three lander.

For this example case, the fixed step, Runge-Kutta integration routine was used with output printed every 10 integration steps. An integration time step of 0.0001 sec with a total real time of 0.1 sec was requested. In addition to the normal printed output, the center body time histories and landing gear loads were output on magnetic tape for later use with the Structural Analysis Program. The retrieval of this information from the tape is discussed in Appendix E. Flexible center body input data was determined with the Structural Analysis Program, as discussed in Sections 4.3.1.3 and 6.2.1.

The lander had three inverted tripod gears with crushable main struts and elastic drag struts. The Secondary Soil Mechanics routine was used in conjunction with a footpad attenuation system. Since the landing case considered was planar in nature, the unnecessary rigid body degrees of freedom were suppressed to conserve computer run time. The equations of motion for all the footpads were integrated during the complete run.

This run, including a flexible center body structure represented with three free-free modes, and the real time of 0.1 sec required 53 sec of CDC 6600 CP time. Computer runs for this length of real time required approximately 34 CP sec to run a case with a rigid center body.

Plots showing various time histories obtained from this run are given in Section 6.2.2. Also shown is a comparison of the results of this run with those obtained with a rigid center body.

6. DEMONSTRATION OF PROGRAM CAPABILITIES

Examples of the capabilities of the various developed computer programs are presented in the following sections. The Task Order Three lander configuration, shown in Figure 1-1, was chosen for many of these analyses since experimental data was available with which to compare the computer results. In general, the correlation between experimental data and the analytical studies was quite good. In addition to the planar landings of the Task Order Three Lander, results for spatial landings on different types of soil are discussed.

6.1 Large Displacement Analysis of Landing Gears - The Landing Gear Option of the Structural Analysis Program was employed to analyze typical inverted tripod and cantilever landing gear configurations. An analysis of the Task Order Three inverted tripod gear to determine drag strut loads versus stroke normal to a landing surface is presented. Predicted drag strut loads are correlated with loads obtained in the sixth drop test on the Task Order Three landing gear performed at NASA Langley Research Center. In addition, drag strut loads as a function of stroke are predicted for an arbitrarily oriented friction plane to indicate the influence on these loads of a non-symmetrically loaded gear.

Analysis of the one-sixth scale LM cantilever gear to determine maximum energy absorption capability for various orientations of the landing surface is discussed in Section 6.1.2. Friction coefficient of the surface was varied to show the way in which friction affects energy absorption.

The fixed finite element idealization of inverted tripod and cantilever landing gears, employed in the Structural Analysis Program Landing Gear Option, was shown in Figure 4-17.

6.1.1 Inverted Tripod Gear - The inverted tripod gear of the Task Order Three Lander was analyzed to determine drag strut loads as a function of gear stroke. A listing of input information and output data for this problem is presented in Appendix C. Only selected pages of computer output are included due to the volume of output. Header card (see Figure 4-36) for this data case does not appear in the output listing although it was the first card of the data set.

Main strut honeycomb properties used for the gear are given in Figure 6-1. The Surface Coordinate System and Lander Coordinate System were assumed to be coincident, with the X_L and X_S axes normal to (and pointing out of) the landing surface (see Figure 4-20). Hence, the Euler angles ψ , θ , and ϕ are equal to 0. This condition corresponds to a "straight-in" drop of a lander such that the drag struts of a gear are loaded symmetrically. Behavior of the gear was investigated in 60 steps for an applied displacement of 15.24 cm normal to the landing surface. Friction coefficient of the surface was assumed to be .8. Footpad honeycomb crush forces of 1.102×10^9 dynes and 4.18×10^9 dynes and corresponding crush strokes of 1.27 cm and 4.45 cm were assumed.

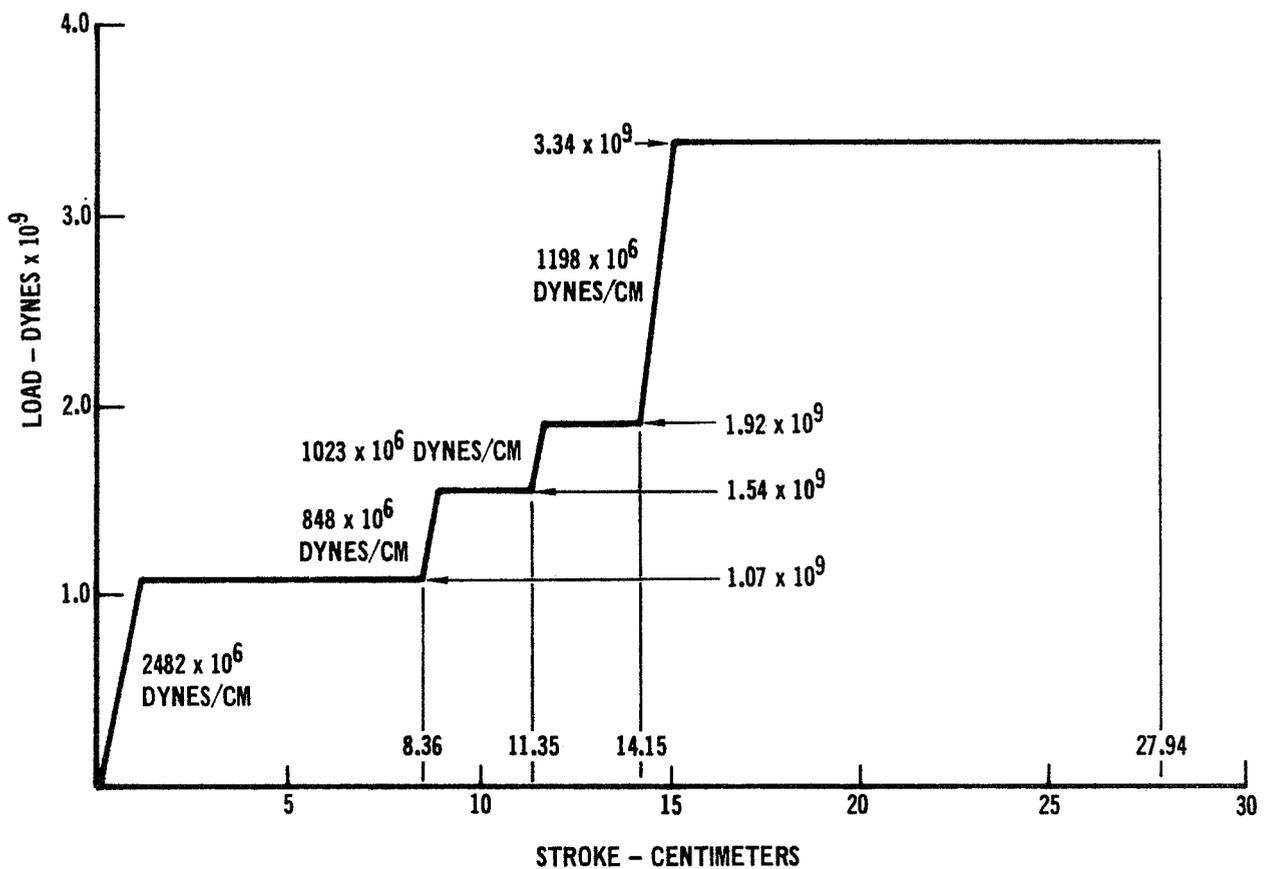


FIGURE 6-1 LOAD - STROKE CURVE FOR MAIN STRUT OF INVERTED TRIPOD GEAR - TASK ORDER THREE LANDER

A correlation between drag strut loads obtained in the computer run (steps 1, 2, 9, 10, 59 and 60 shown in Appendix C) and loads obtained for this gear during the sixth leg drop test performed at NASA Langley Research Center is depicted in Figure 6-2. As illustrated, at the onset of stroking the experimental drag strut loads appear to build up instantaneously. This is due to large footpad inertia forces acting during initial footpad impact; thus, correlation of drag strut loads at the onset of stroking is poor since the program assumes that the struts are initially unloaded. Footpad inertia forces are negligible after a normal (X_S) stroke of about 1 cm and from this point on correlation is reasonably good. In the figure, only one curve is shown for predicted drag strut loads since the drag struts were loaded equally because of symmetry. This example indicates that the static analysis employed in the Landing Gear Option can be used to obtain an estimate of dynamic drag strut loads.

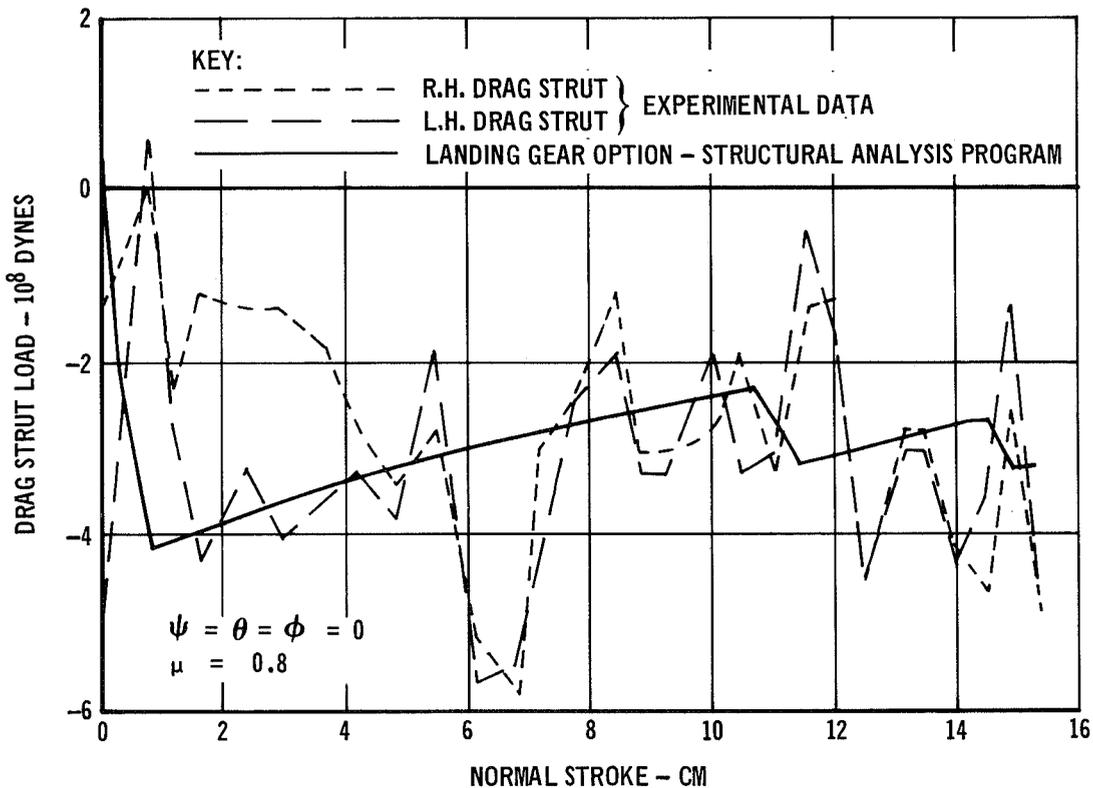


FIGURE 6-2 DRAG STRUT LOAD CORRELATION - INVERTED TRIPOD GEAR

For a nonsymmetrically loaded gear, the drag strut loads may be considerably higher than for the symmetrically loaded case. To emphasize this fact, the inverted tripod gear described above was analyzed with the program using the same input data with the exception that the landing surface was oriented such that $\psi = 25^\circ$ and $\theta = \phi = 0$. This condition can be visualized by referring to Figure 4-20 and imagining that the landing surface is rotated 25° about the Z_G direction. For this run, drag strut loads as a function of stroke normal to the rotated landing surface are shown in Figure 6-3. As shown, one drag strut is in tension while the other is in compression. The maximum drag strut load of 4.52×10^9 dynes is considerably higher than the drag strut loads shown in Figure 6-2 for the symmetrically loaded gear.

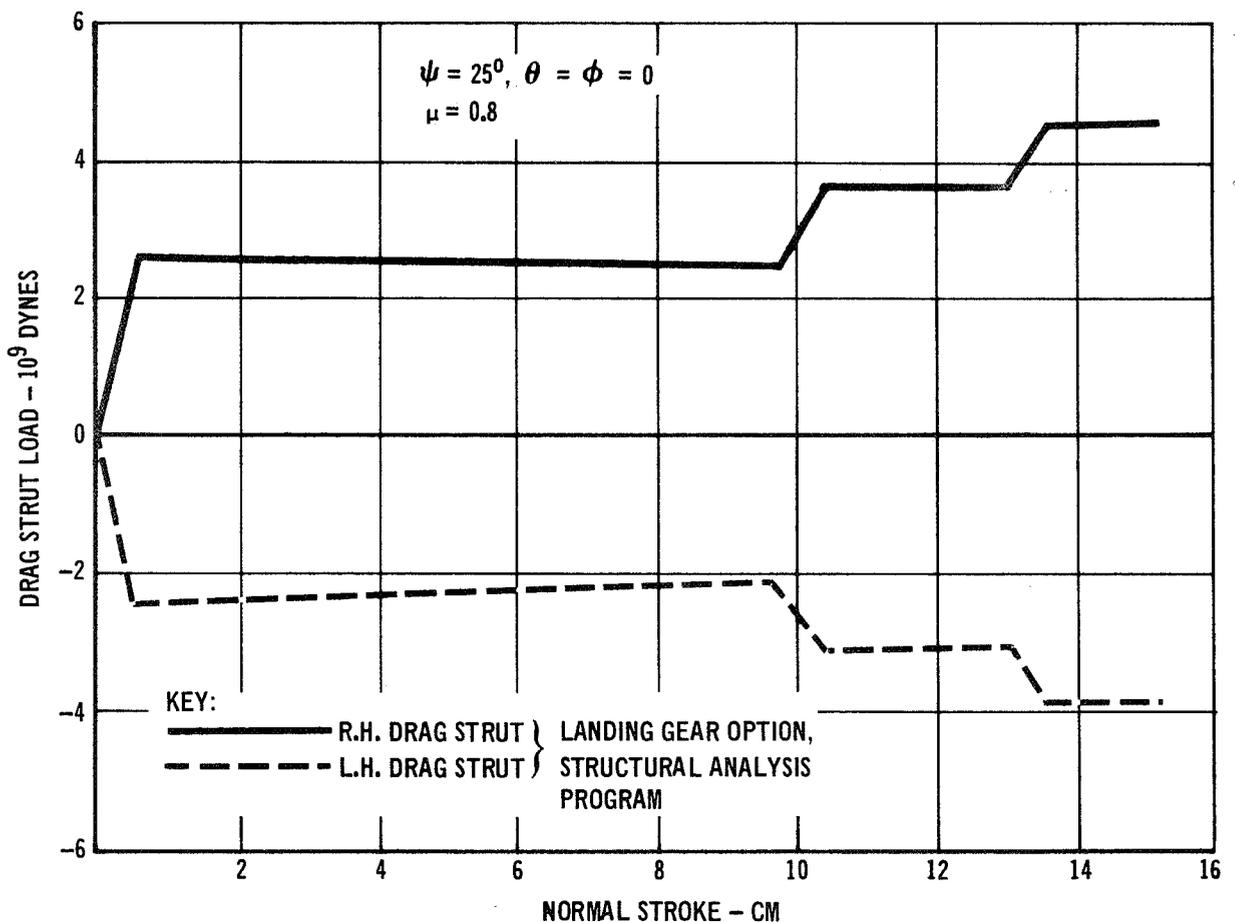


FIGURE 6-3 ANTICIPATED DRAG STRUT LOADS FOR INVERTED TRIPOD GEAR

6.1.2 Cantilever Gear - The one-sixth scale LM cantilever gear was analyzed to determine maximum energy absorption capability as a function of landing surface orientation and friction coefficient.

Honeycomb crush properties assumed for the main strut and drag struts of the gear are shown in Figures 6-4 and 6-5. Various orientations of the landing surface (see Figure 4-20) were considered, each of which provided symmetrical loading of the drag struts. These orientations are described by the Euler angle combinations $\psi = 0^\circ$, $\phi = 0^\circ$, and θ varied from 0° to 40° . The maximum value of 40° is the sum of the maximum ground slope relative to the local horizontal (30°) and the maximum pitch-up of the lander (10°) specified in typical design constraints, Section 3.1. Behavior of the gear was investigated by applying a displacement of 30.48 cm, normal to the landing surface, in 100 steps. This large value of normal displacement was selected to insure that the gear bottomed out before all steps were taken. For each coefficient of friction considered, a series of computer runs, were made in which θ was varied from 0° to 40° . Friction coefficients of 0.2, 0.8, and 1.0 were considered.

Maximum energy absorbed by the gear in a direction normal to the landing surface (X_S direction) is presented in Figure 6-6. Each point in the figure represents the total energy absorbed by the gear before the main strut bottomed out. Curves for friction coefficients of 0.8 and 1.0 are the same because in each case the friction was sufficient to prevent the footpad from sliding. For the friction coefficient of 0.2, the curve includes the effect of footpad sliding on the landing surface. As evidenced in this figure, energy absorption capability of the gear is minimum for values of θ between 20° and 30° .

6.2 Analysis of Task Order Three Lander - A complete analysis of a typical legged lander configuration is discussed in the following sections. This study consisted of performing a modal analysis on the center body structure employing the Center Body Option of the Structural Analysis Program. This modal data was then incorporated in the Landing Loads and Motions Program to account for the effects of a flexible center body on the landing response of the vehicle. Output from the landing program was used, in conjunction with the Center Body Landing Loads Program, to determine the distribution of inertia, gravity, and landing gear strut loads throughout the center body. These loads

were then input to the Center Body Option of the Structural Analysis Program to determine internal member load distributions.

The legged lander considered was the 3/8 mass version of the Task Order Three lander. Figure 6-7 shows the structural configuration of this vehicle. Included in following analysis of this lander are examples of correlation between predicted results obtained with the Landing Loads and Motions Program and the results of the third drop test conducted at NASA Langley Research Center.

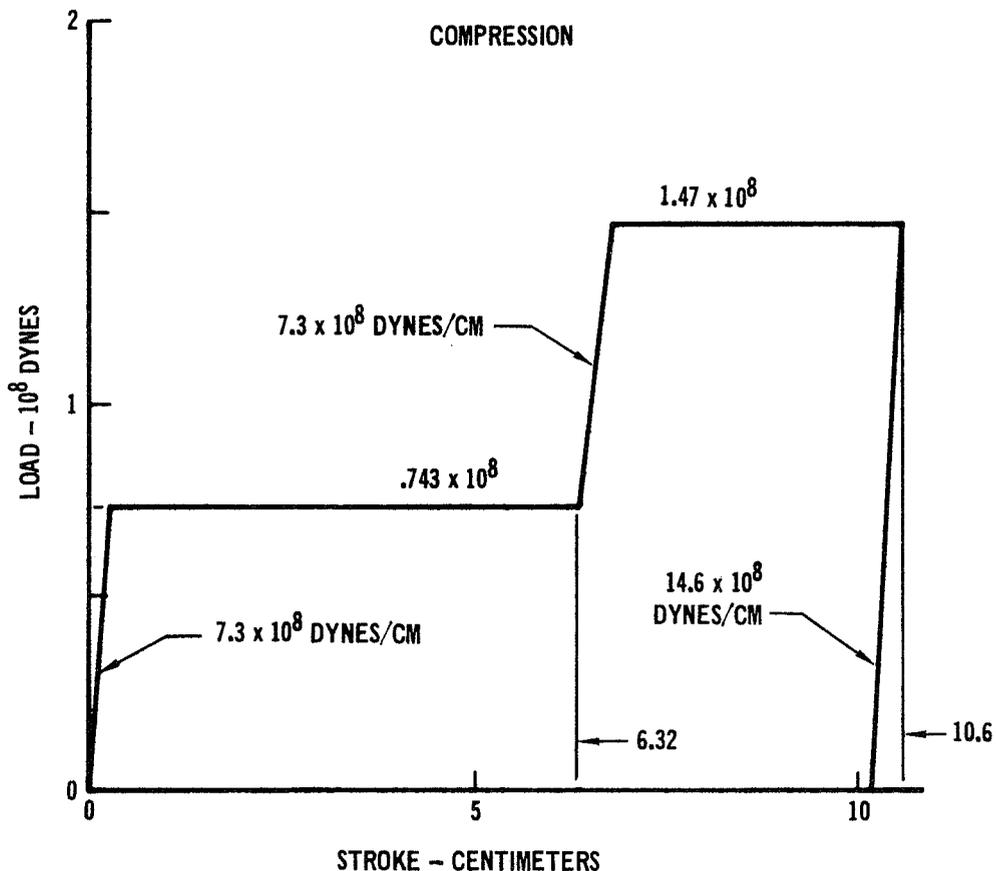


FIGURE 6-4 LOAD-STROKE CURVE FOR MAIN STRUT OF ONE-SIXTH SCALE LM CANTILEVER GEAR

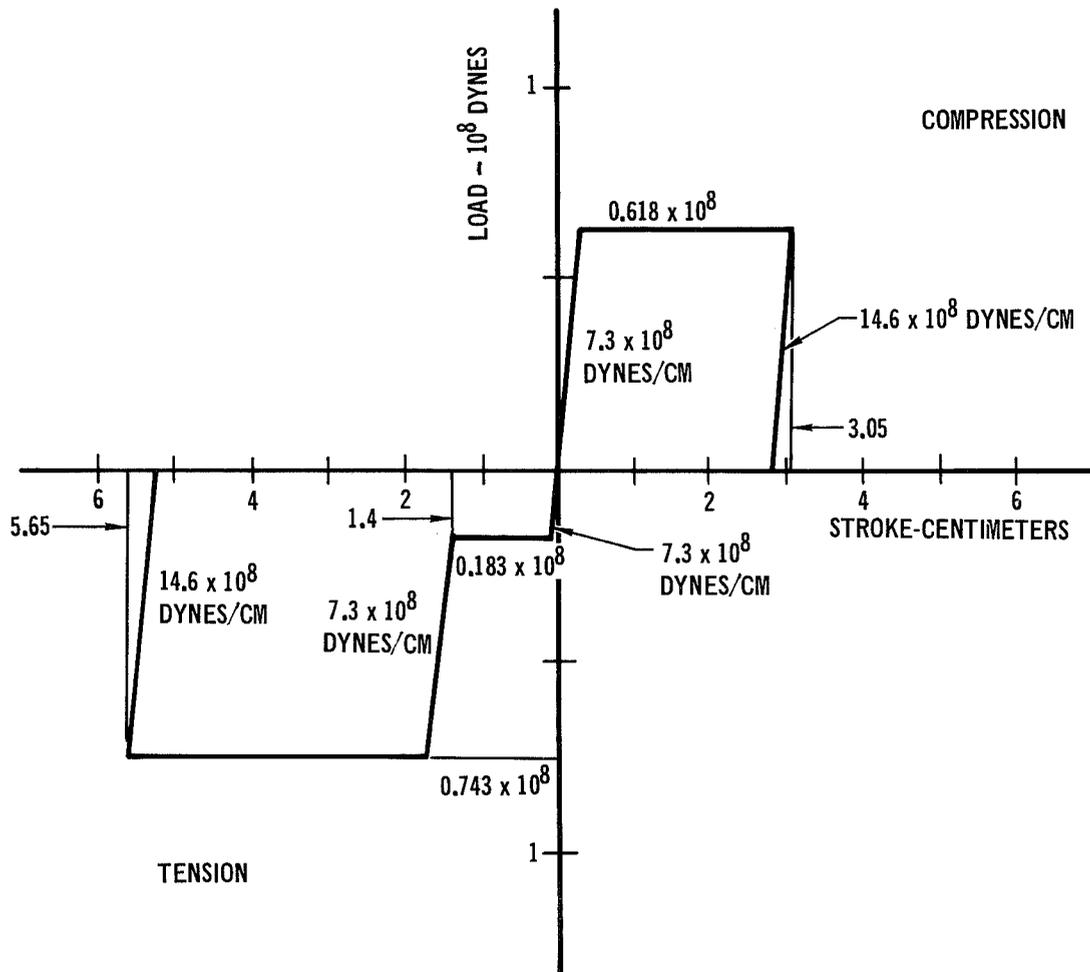


FIGURE 6-5 LOAD-STROKE CURVE FOR DRAG STRUT OF ONE-SIXTH SCALE LM CANTILEVER GEAR

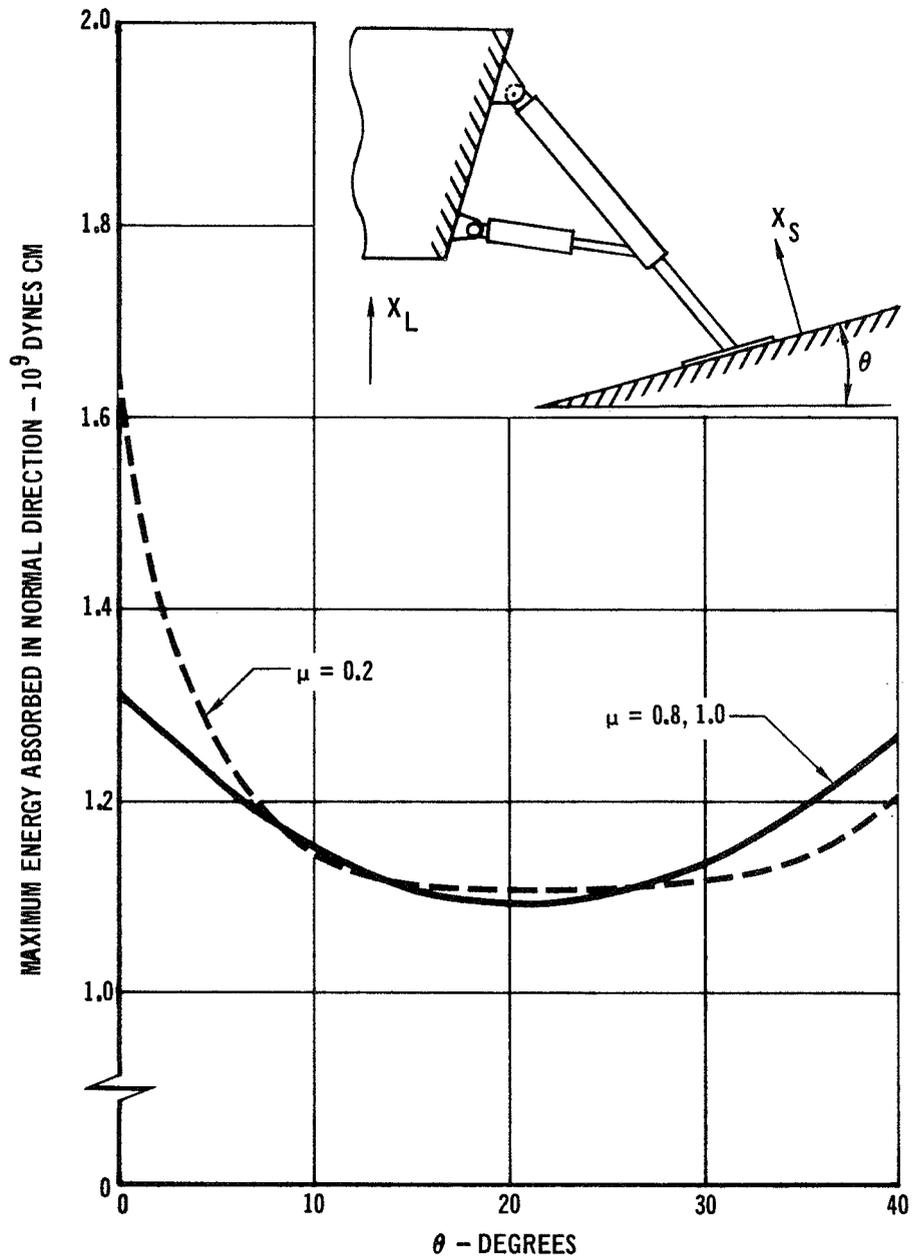


FIGURE 6-6 MAXIMUM ENERGY ABSORPTION FOR ONE-SIXTH SCALE LM CANTILEVER GEAR

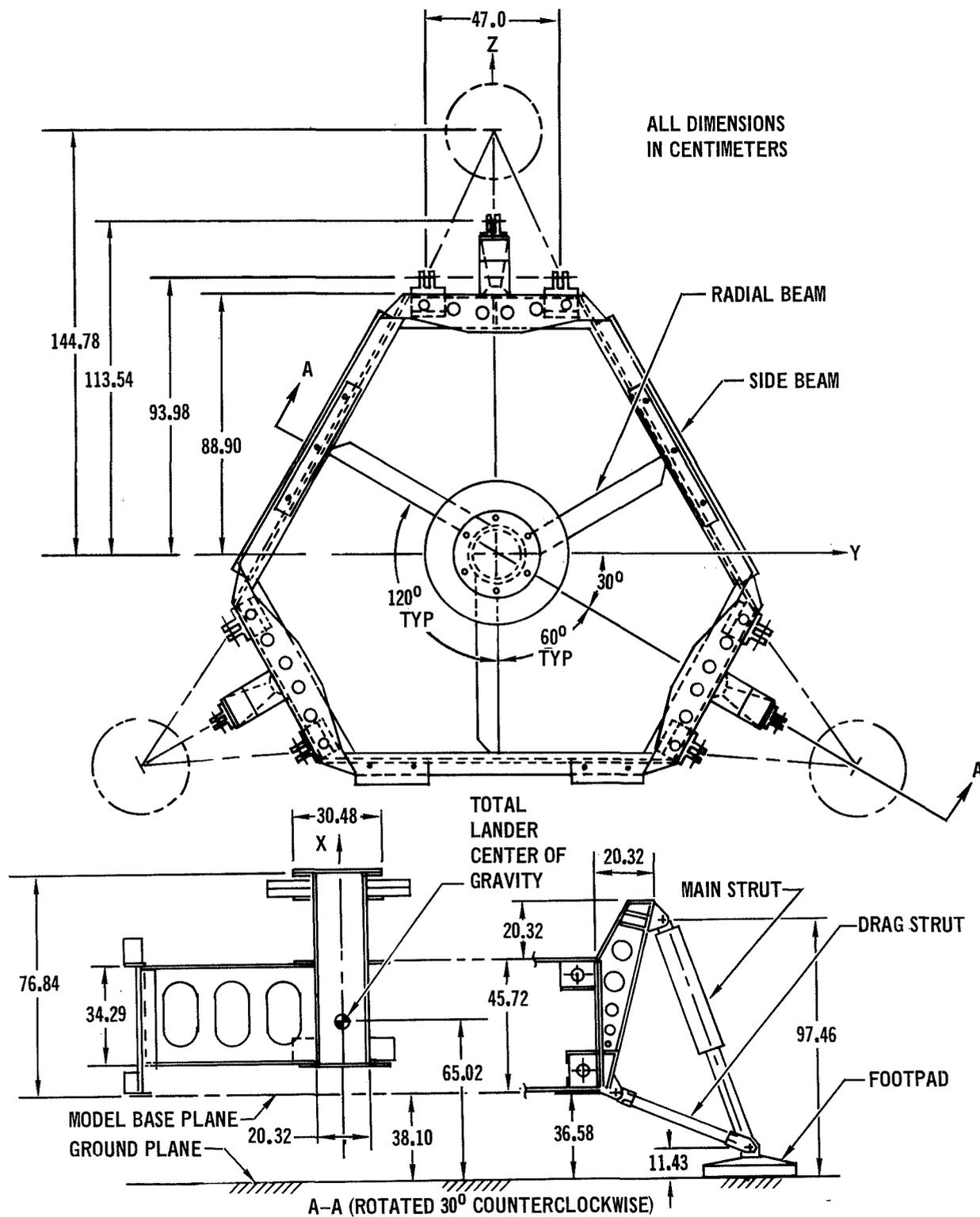


FIGURE 6-7 GEOMETRY OF TASK ORDER THREE LANDER

6.2.1 Modal Analysis of Center Body Structure - The finite element idealization of the center body structure is shown in Figure 6-8. The member stiffness properties for each pair of bars representing a side beam (for example, members 10 and 11) and a radial beam (members 48 and 49) are such that the total stiffness of these deep beams is maintained. The moment carrying capabilities of members 46, 47, 48, 49, 50, and 51 about the lander X axis, where these members attach to the side beams, were removed. This was done to simulate the high flexibility associated with these attach points.

This idealization resulted in a stiffness matrix containing 33 joints with 198 degrees of freedom. Before obtaining the structure's frequencies and mode shapes, the stiffness matrix was reduced to 99 degrees of freedom by removing the three rotational degrees of freedom at each joint. Thus, the input mass matrix contained translational masses associated with all of the joints shown in Figure 6-8.

Twenty elastic modes were requested and the program output for this modal analysis is shown in Appendix B. Three modes, with frequencies of 34.6 Hz, 48.2 Hz, and 101 Hz were obtained whose modal deformation patterns were predominately in the lander Y-Z plane and are the type of modes that would be excited by a landing such as the third drop test of the Task Order Three lander. These are the modes which were included in the data for the Landing Loads and Motions Program, Section 6.2.2, when considering the effects of a flexible center body structure. Figures 6-9 through 6-11 show the mode shapes associated with these three frequencies.

6.2.2 Correlation with Third Drop Test - The predicted landing response for the initial conditions corresponding to NASA Langley's third drop test were obtained considering both a rigid and flexible center body structure. The initial conditions for this drop test condition are shown in Figure 6-12. A portion of the output from the Landing Loads and Motions Program for the flexible body is presented in Appendix D.

The main strut load-stroke relationship shown in Figure 6-1 was used for these correlation runs. A sketch of the actual footpad geometry, including footpad attenuation, and the footpad idealization employed in the landing analyses are shown in Figure 6-13. The Secondary Soil Mechanics routine was

employed with the following soil parameters:

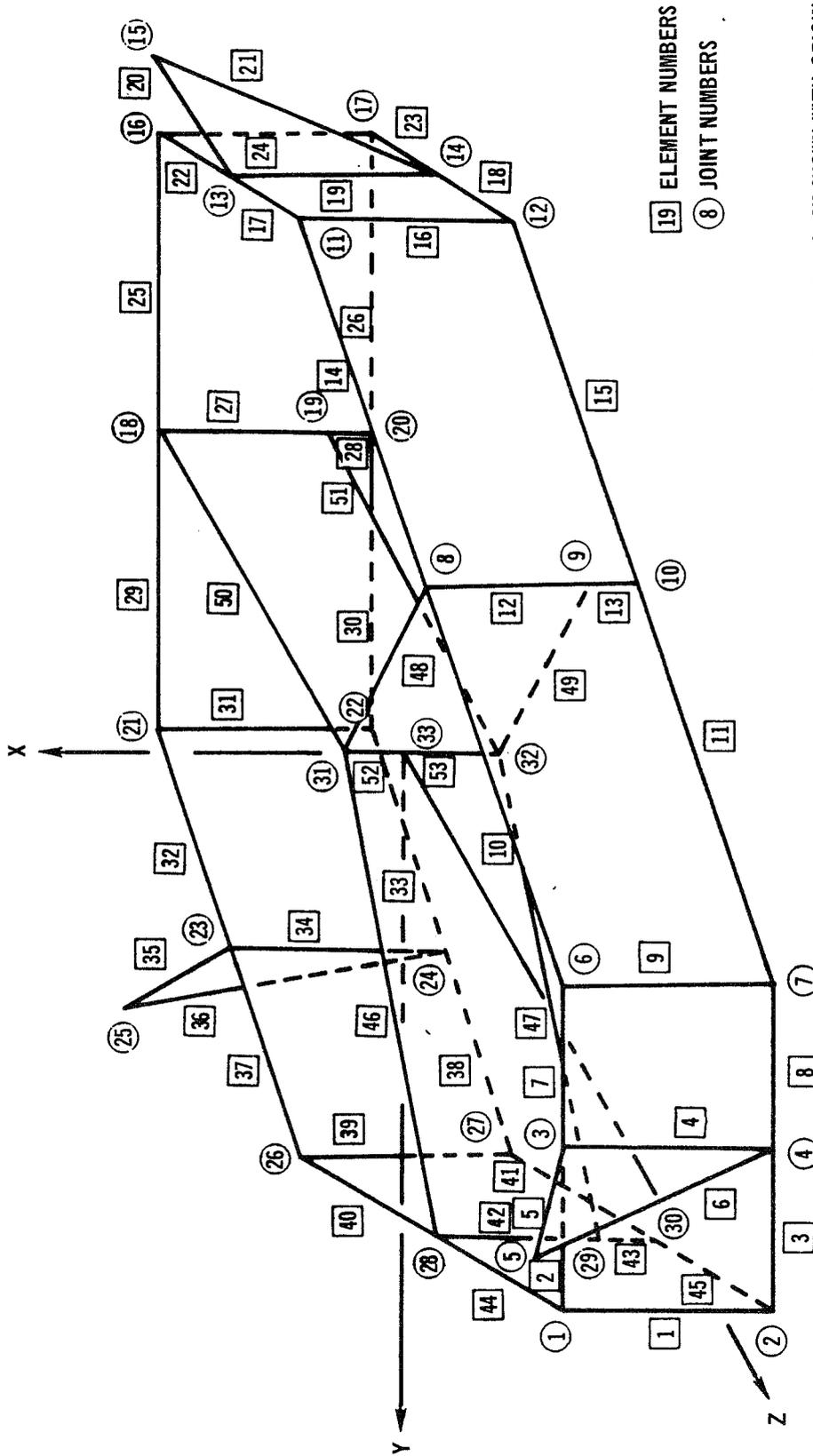
Soil elasticity constant = 1.4×10^7 dynes/cm³
Coefficient of friction = 0.9

The distribution of center body mass and inertia and footpad masses used for the 3/8 mass Task Order Three lander in these runs is summarized in Figure 6-14. This breakdown was obtained by requiring that the combination of the center body mass moments of inertia and the footpad masses, multiplied by the square of their respective moment arms, approximated the moments of inertia of the total 3/8 mass Task Order Three lander. In addition, the center of gravity location for the total lander was maintained with this mass distribution.

Predicted center of gravity acceleration time histories and loads in the outside drag strut of leg 2 for a rigid center body are compared with the experimental data in Figures 6-15 and 6-16. The high frequency oscillations present in both the predicted load and acceleration results are the spring-mass effect of the footpad mass in conjunction with the landing gear struts. In addition, oscillation in the drag strut load near the end of the time history (when the footpad is off the ground) is the gear-mass system experiencing free vibration.

Comparison between measured and predicted center of gravity accelerations, main strut loads, main strut strokes, and drag strut loads when center body flexibility was included, is shown in Figures 6-17 through 6-22. Correlation between analytical and experimental results is good in this case. The inclusion of center body flexibility results in a less rapid buildup of center of gravity accelerations upon initial impact, especially in the Z direction. In addition, the inclusion of structural flexibility can readily be seen by the low frequency oscillation present in the center of gravity Z acceleration.

An interesting point is indicated in Figure 6-22. It can be seen that there is an initial oscillatory load in this drag strut before the footpad impacts the landing surface. This is the result of the inertia loading of the footpad mass on the end of the elastic landing gear struts. This inertia loading effect of the noncontacting footpad is also indicated in Figure 6-18 by the buildup of main strut load in leg 1 before footpad impact.



NOTE: LANDER COORDINATE SYSTEM SHOWN WITH ORIGIN AT CENTER OF GRAVITY OF CENTER BODY.

FIGURE 6-8 CENTER BODY STRUCTURAL IDEALIZATION

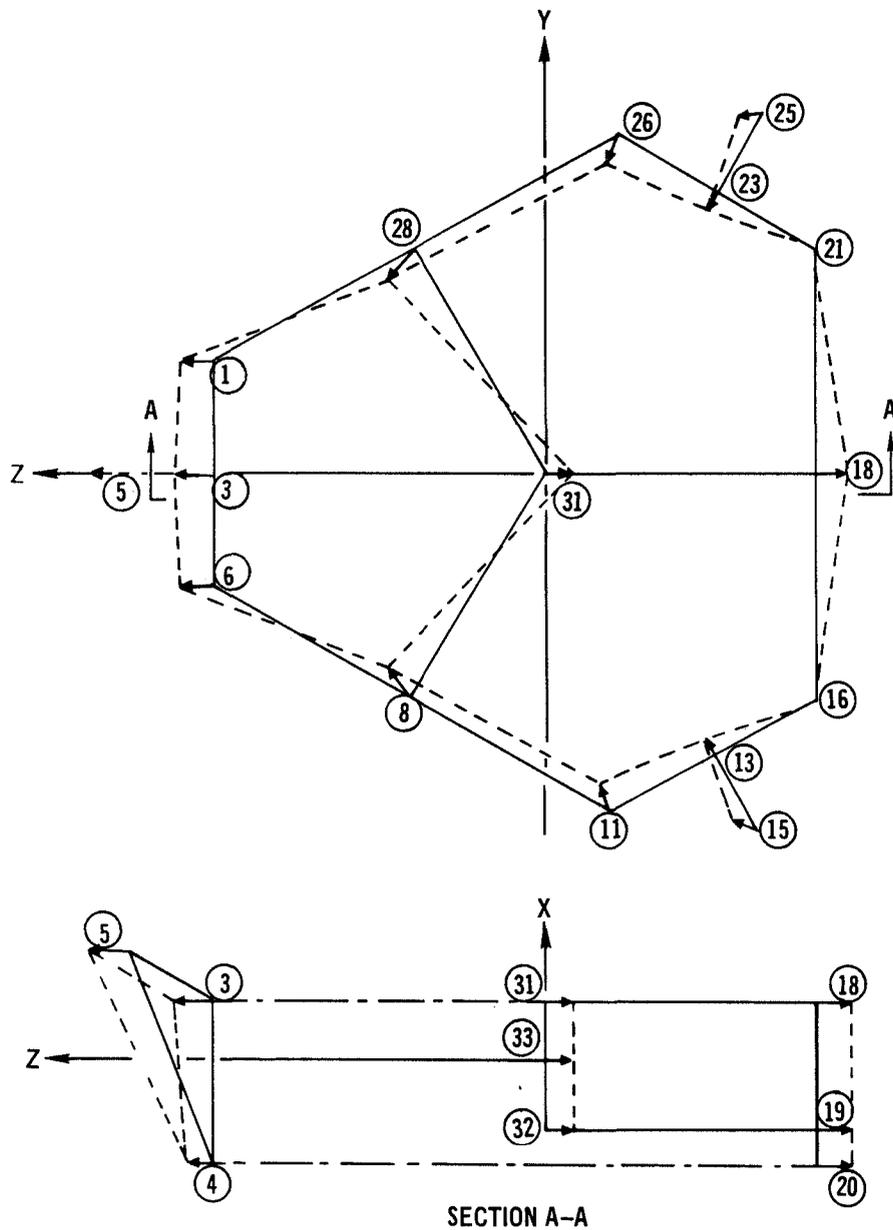


FIGURE 6-9 CENTER BODY MODE SHAPE ASSOCIATED WITH 34.6 H_z MODE

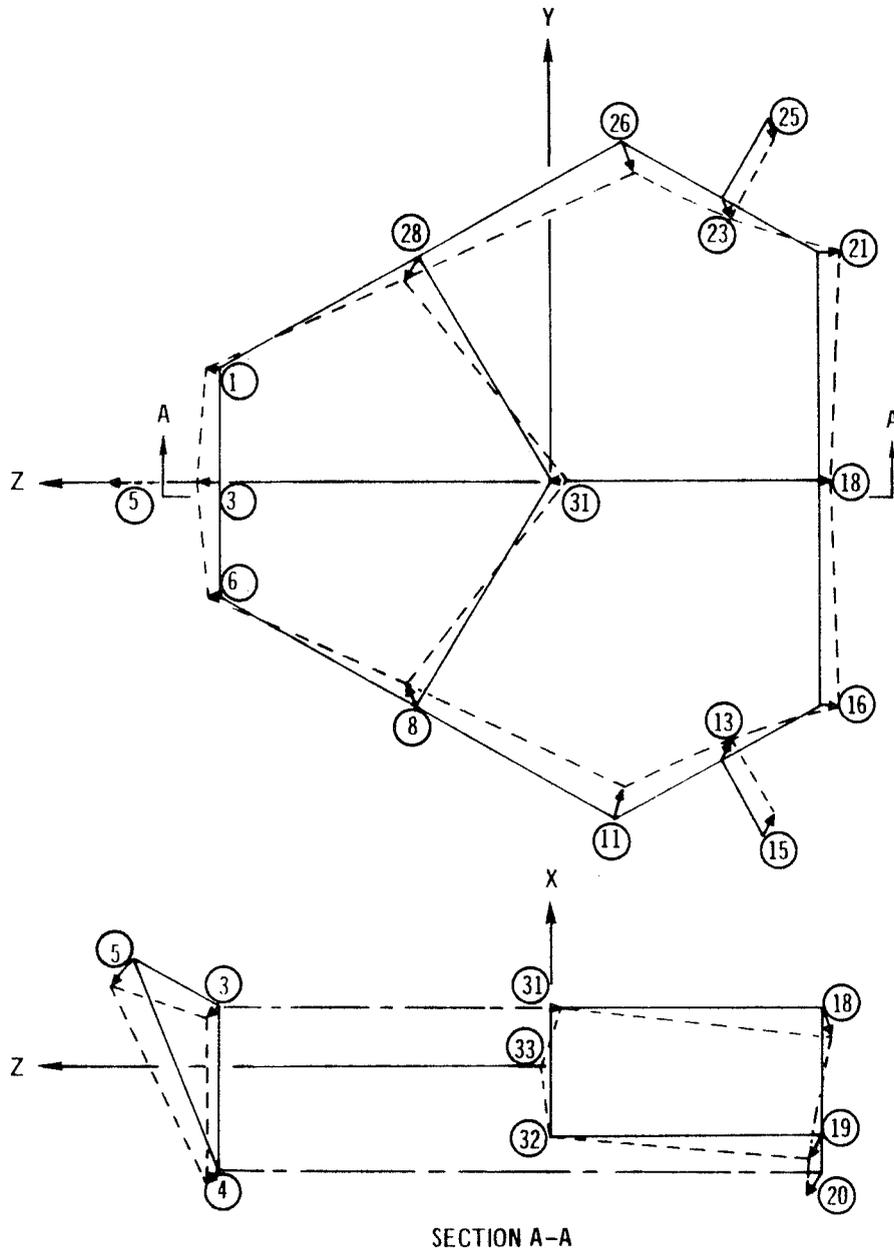


FIGURE 6-10 CENTER BODY MODE SHAPE ASSOCIATED WITH 48.2 H_z MODE

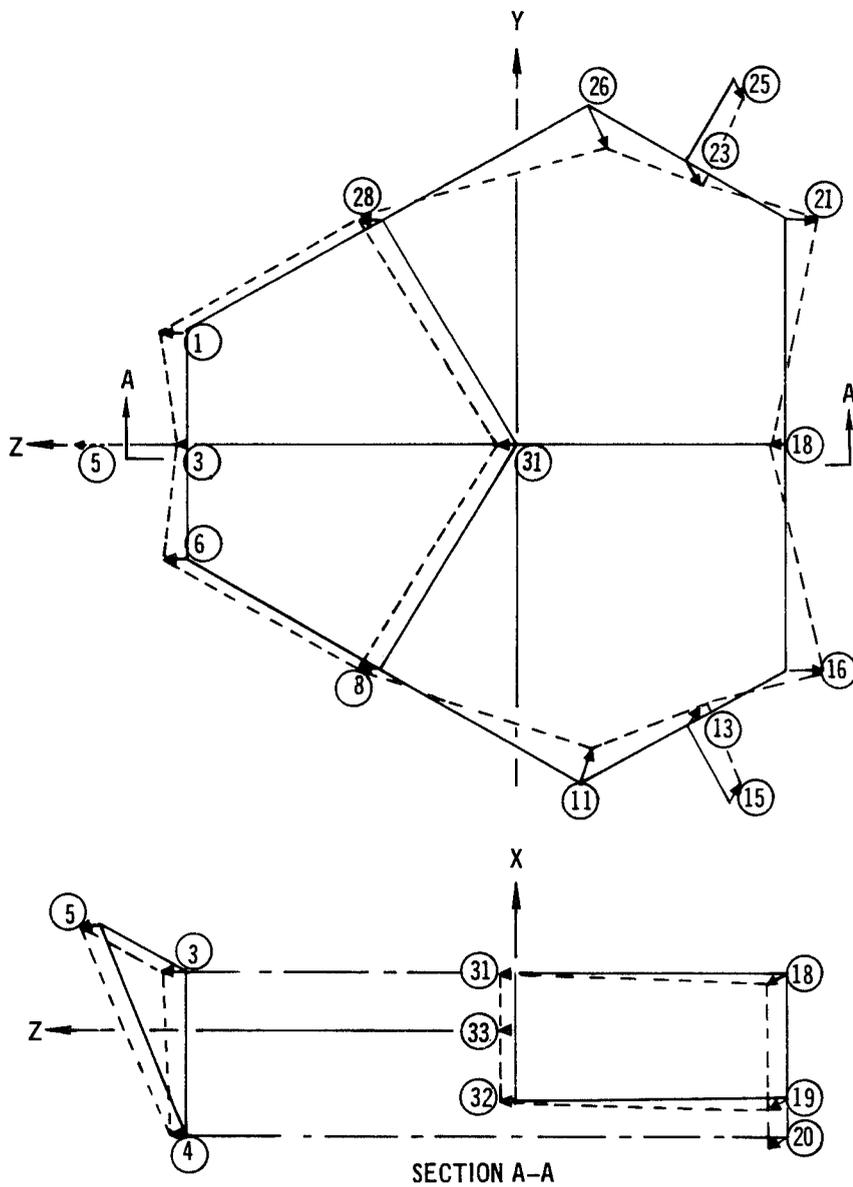


FIGURE 6-11 CENTER BODY MODE SHAPE ASSOCIATED WITH 101 H_z MODE

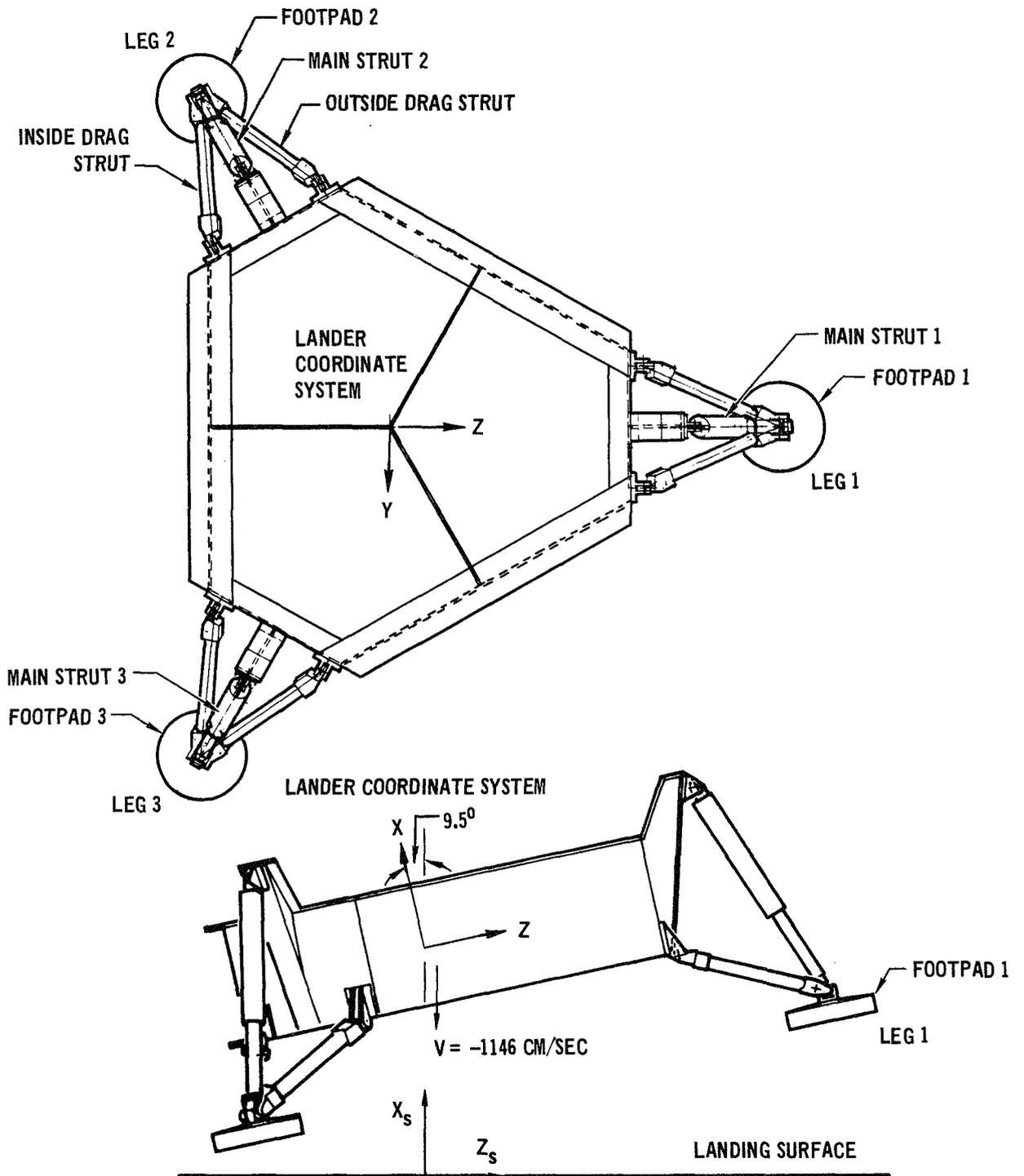
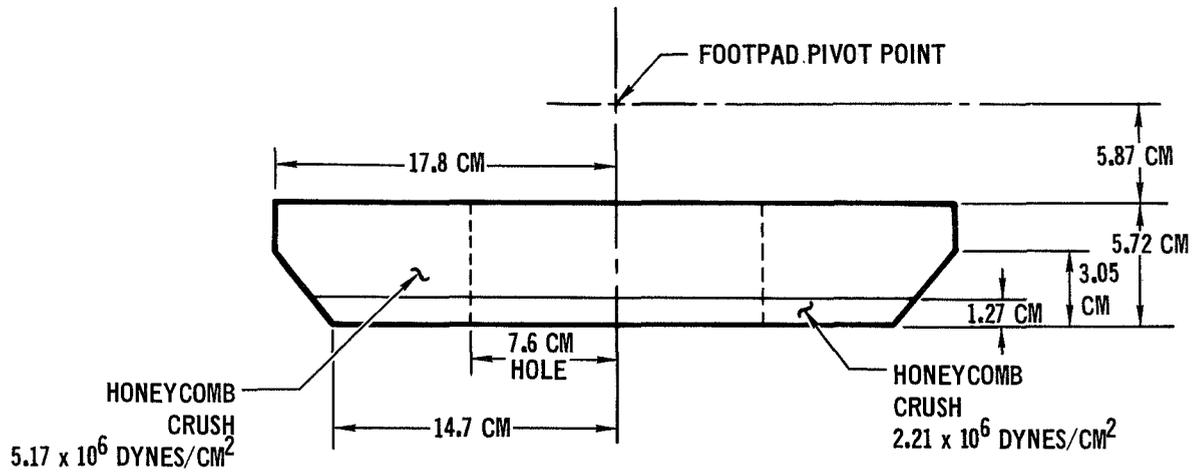
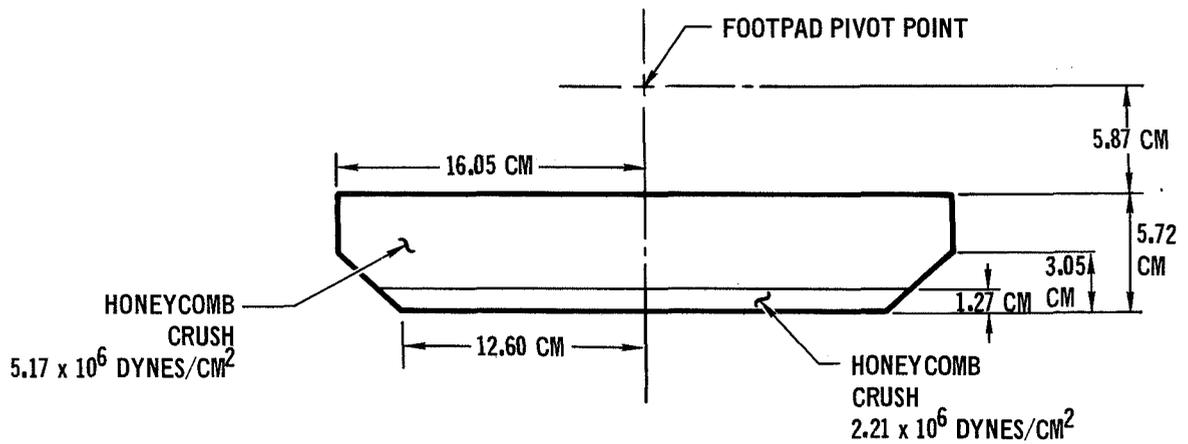


FIGURE 6-12 INITIAL CONDITIONS FOR DROP TEST THREE



ACTUAL FOOTPAD



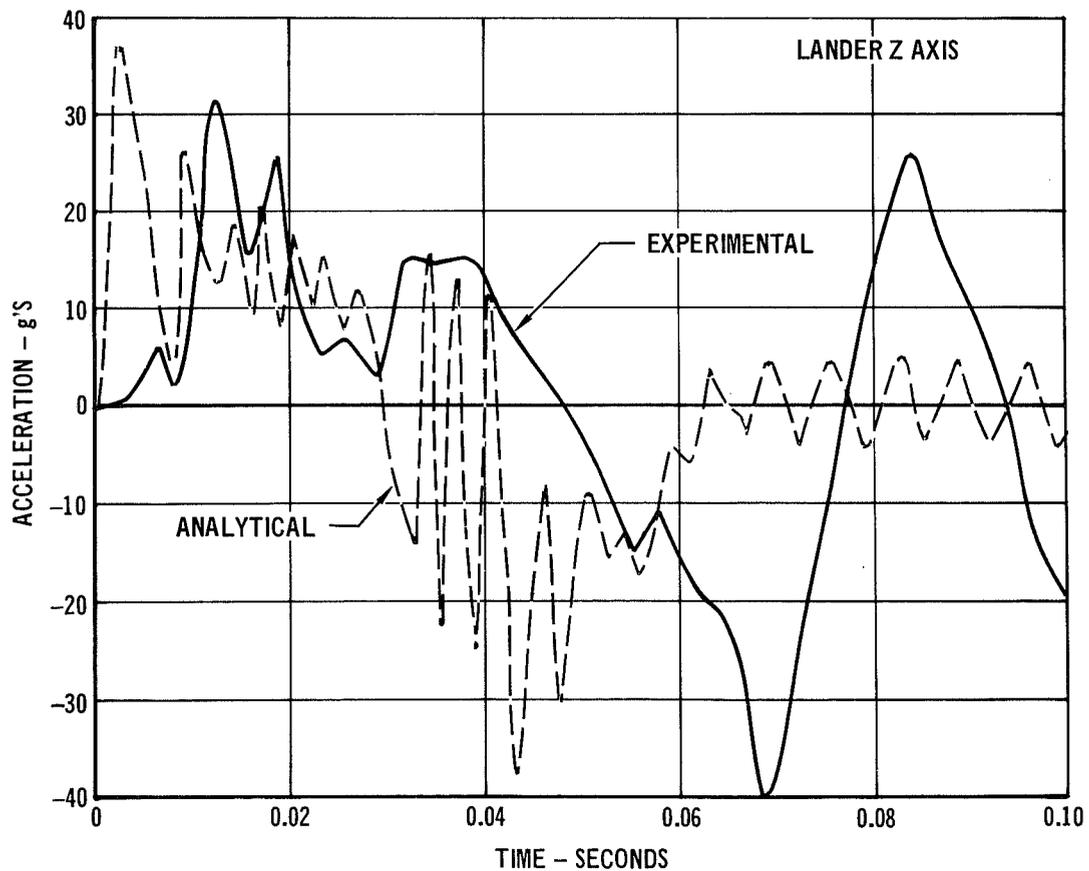
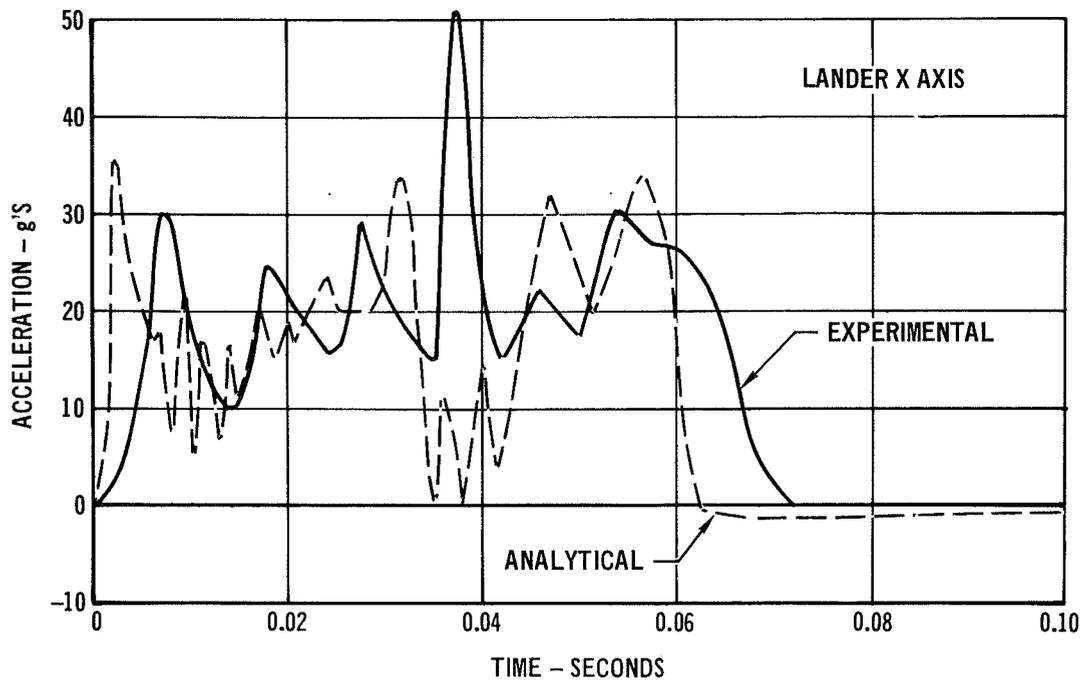
IDEALIZED FOOTPAD

FIGURE 6-13 FOOTPAD IDEALIZATION

ITEM	MASS (GRAMS)	X* (CM)	Y (CM)	Z (CM)	I_{xx} (GRAM-CM ²)	I_{yy} (GRAM-CM ²)	I_{zz} (GRAM-CM ²)
CENTER BODY	180,500	67.97	0	0	8.031×10^8	4.992×10^8	4.968×10^8
FOOTPAD 1	3,497	11.43	0	144.78	-	-	-
FOOTPAD 2	3,497	11.43	-125.37	- 72.39	-	-	-
FOOTPAD 3	3,497	11.43	125.37	- 72.39	-	-	-

* RELATIVE TO BOTTOM OF FOOTPAD

FIGURE 6-14 3/8 MASS TASK ORDER THREE LANDER MASS BREAKDOWN



**FIGURE 6-15 CENTER OF GRAVITY ACCELERATIONS
WITH RIGID CENTER BODY**

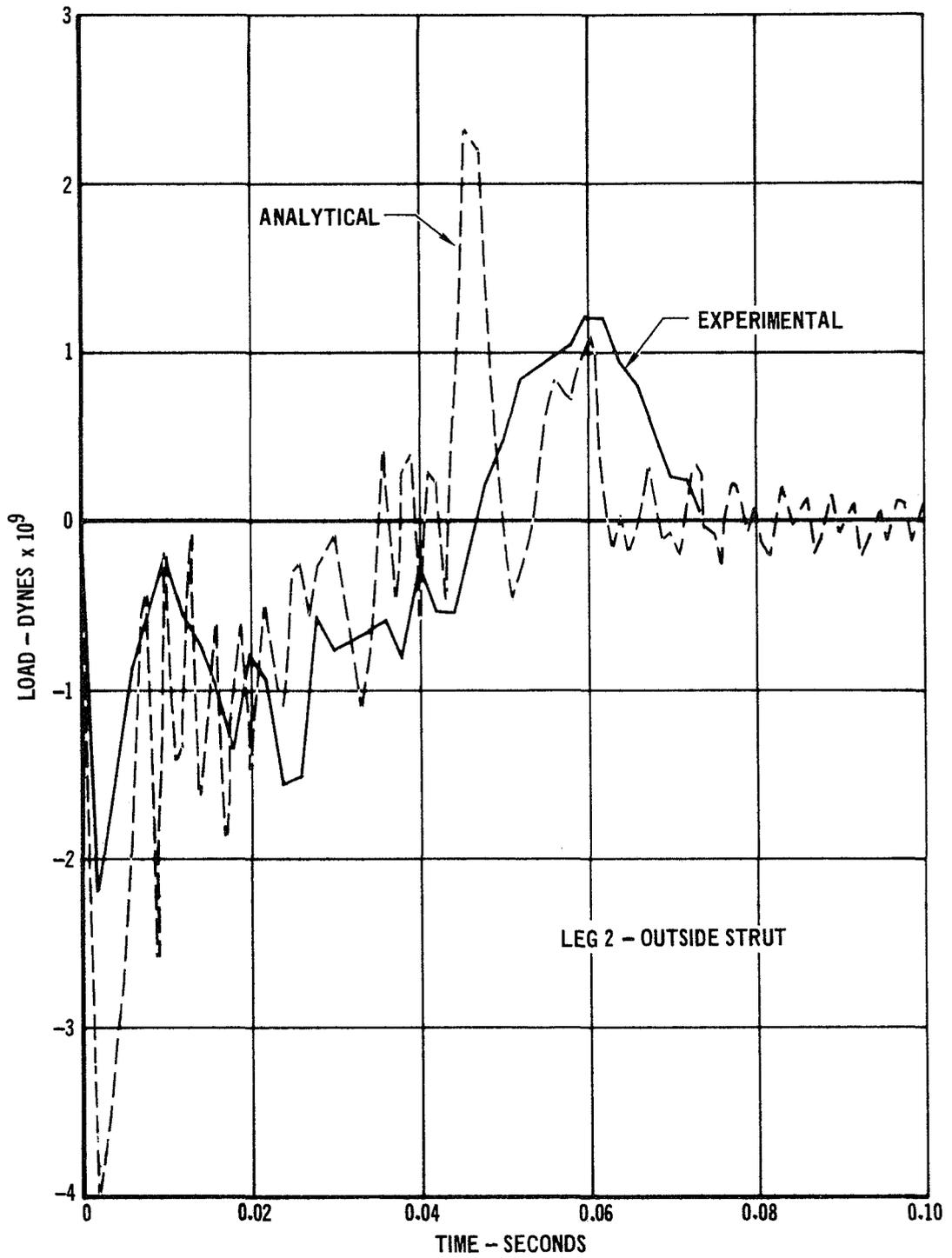


FIGURE 6-16 OUTSIDE DRAG STRUT LOAD WITH RIGID CENTER BODY

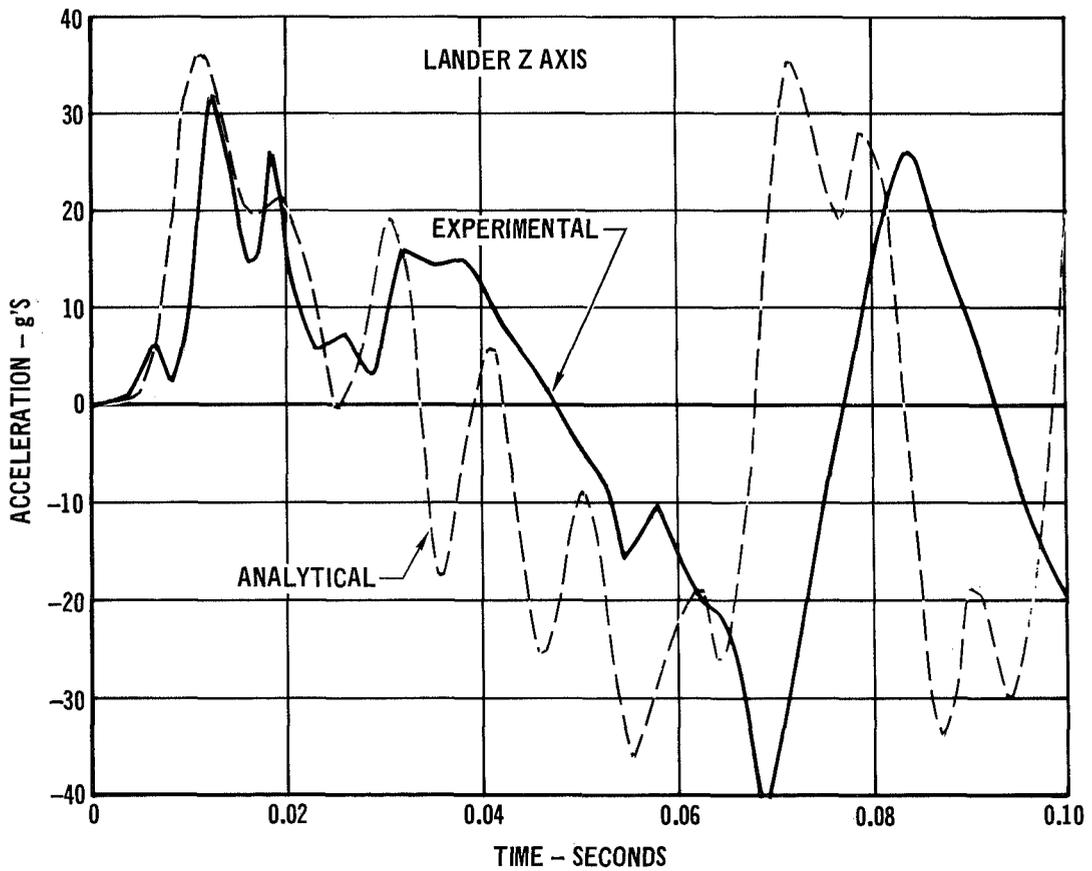
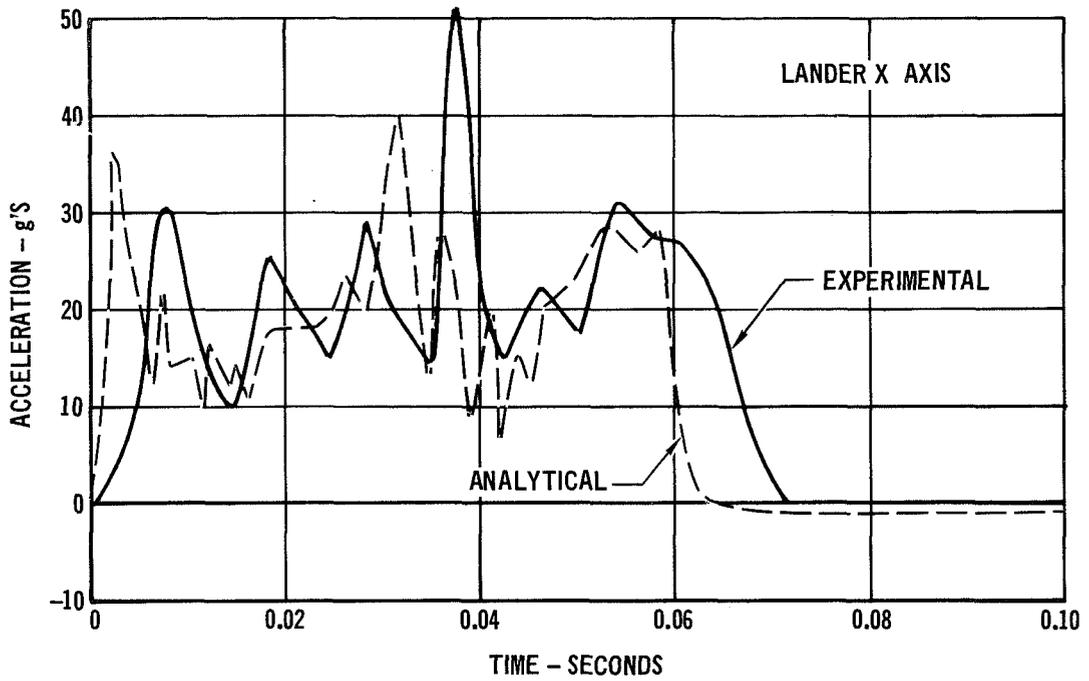


FIGURE 6-17 CENTER OF GRAVITY ACCELERATIONS WITH FLEXIBLE CENTER BODY

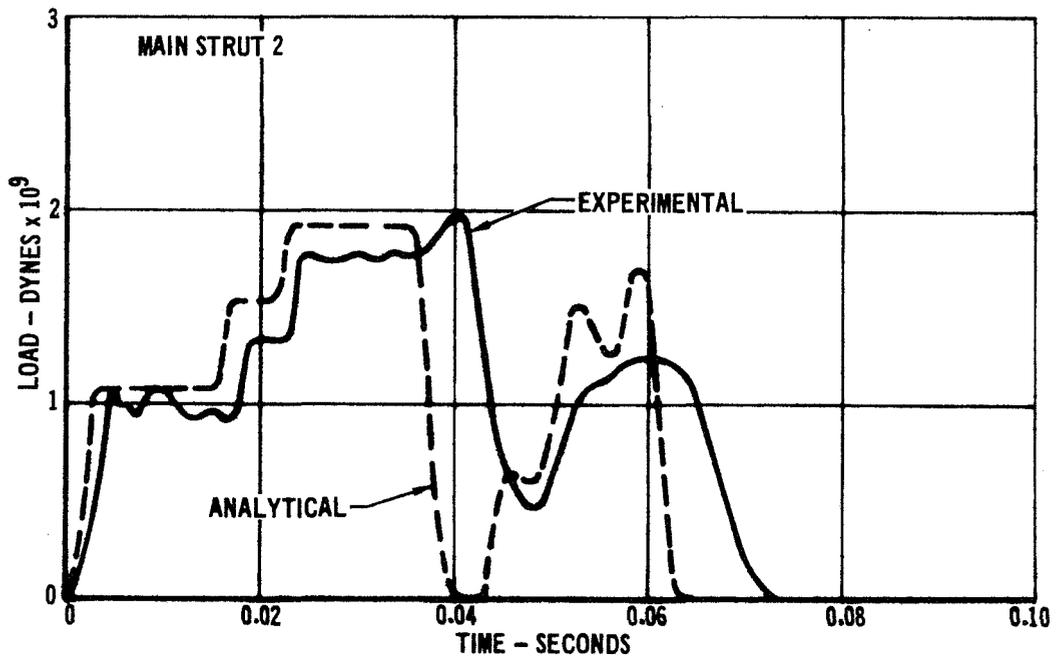
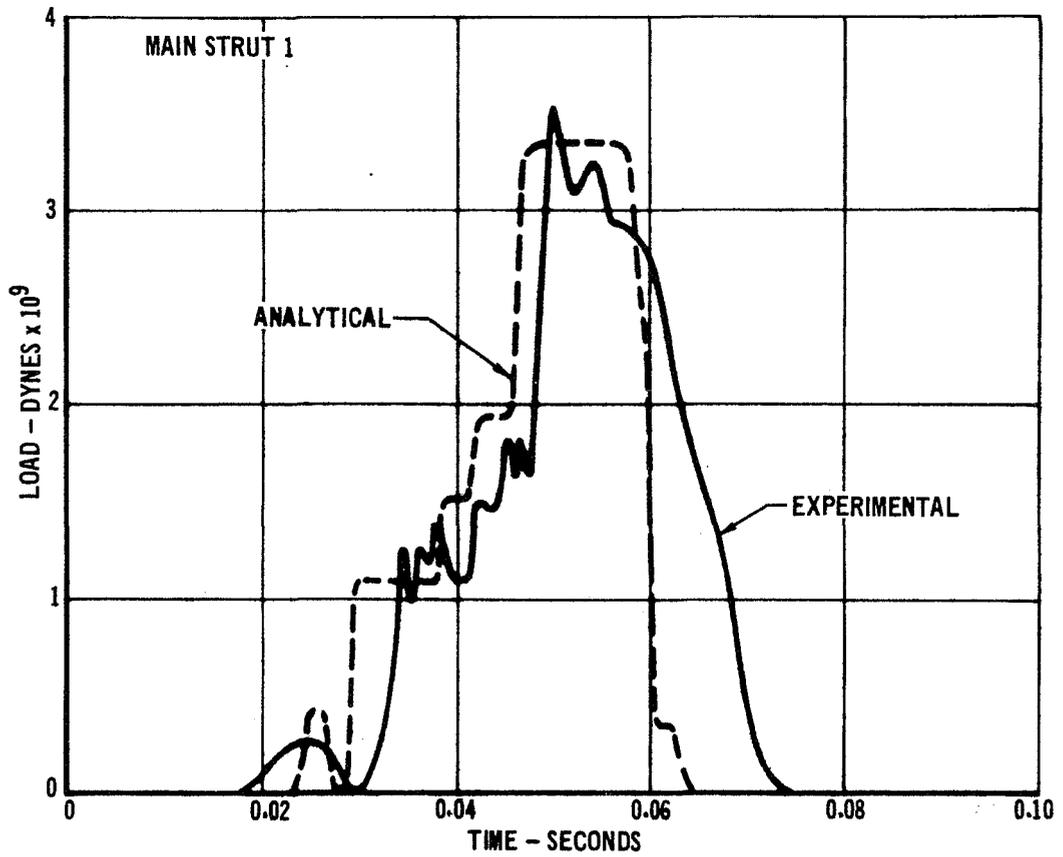


FIGURE 6-18 MAIN STRUT LOADS WITH FLEXIBLE CENTER BODY

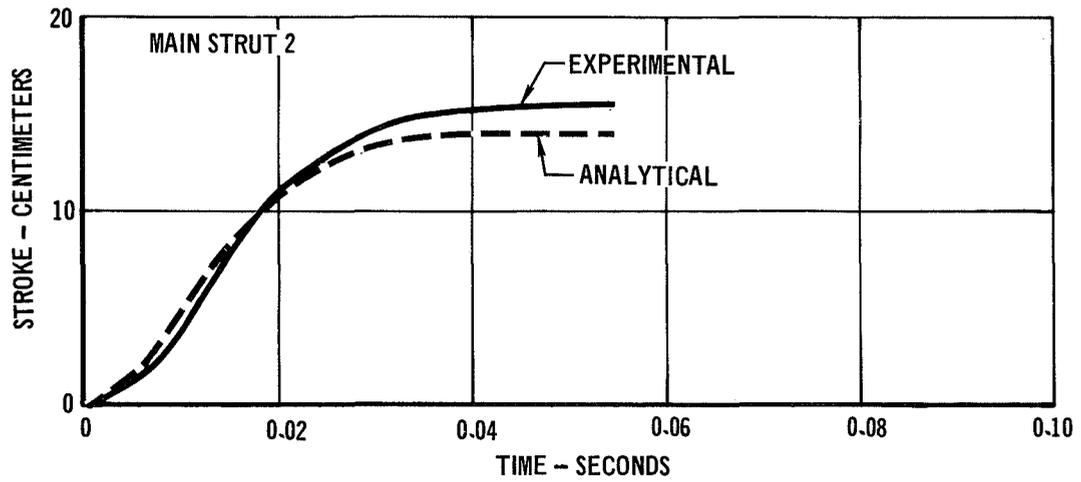
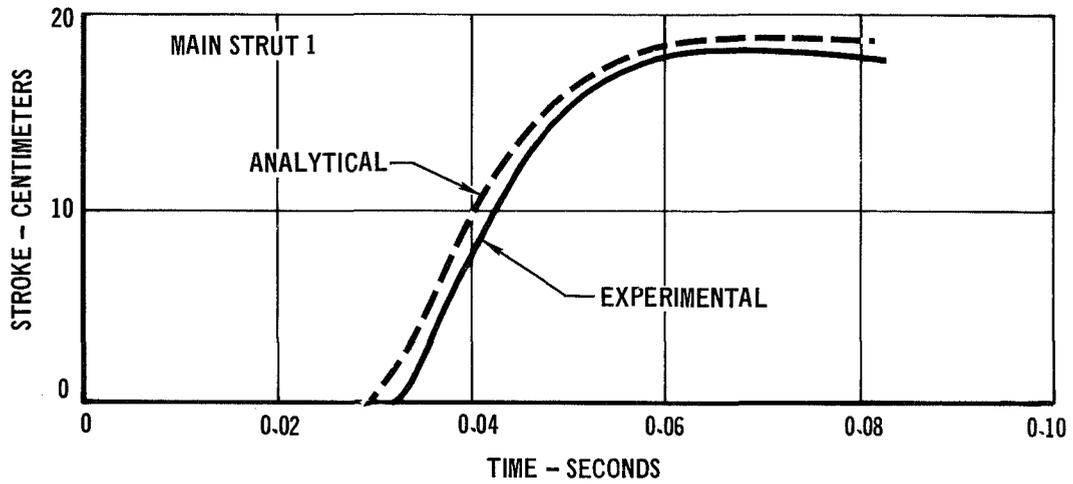


FIGURE 6-19 MAIN STRUT STROKES WITH FLEXIBLE CENTER BODY

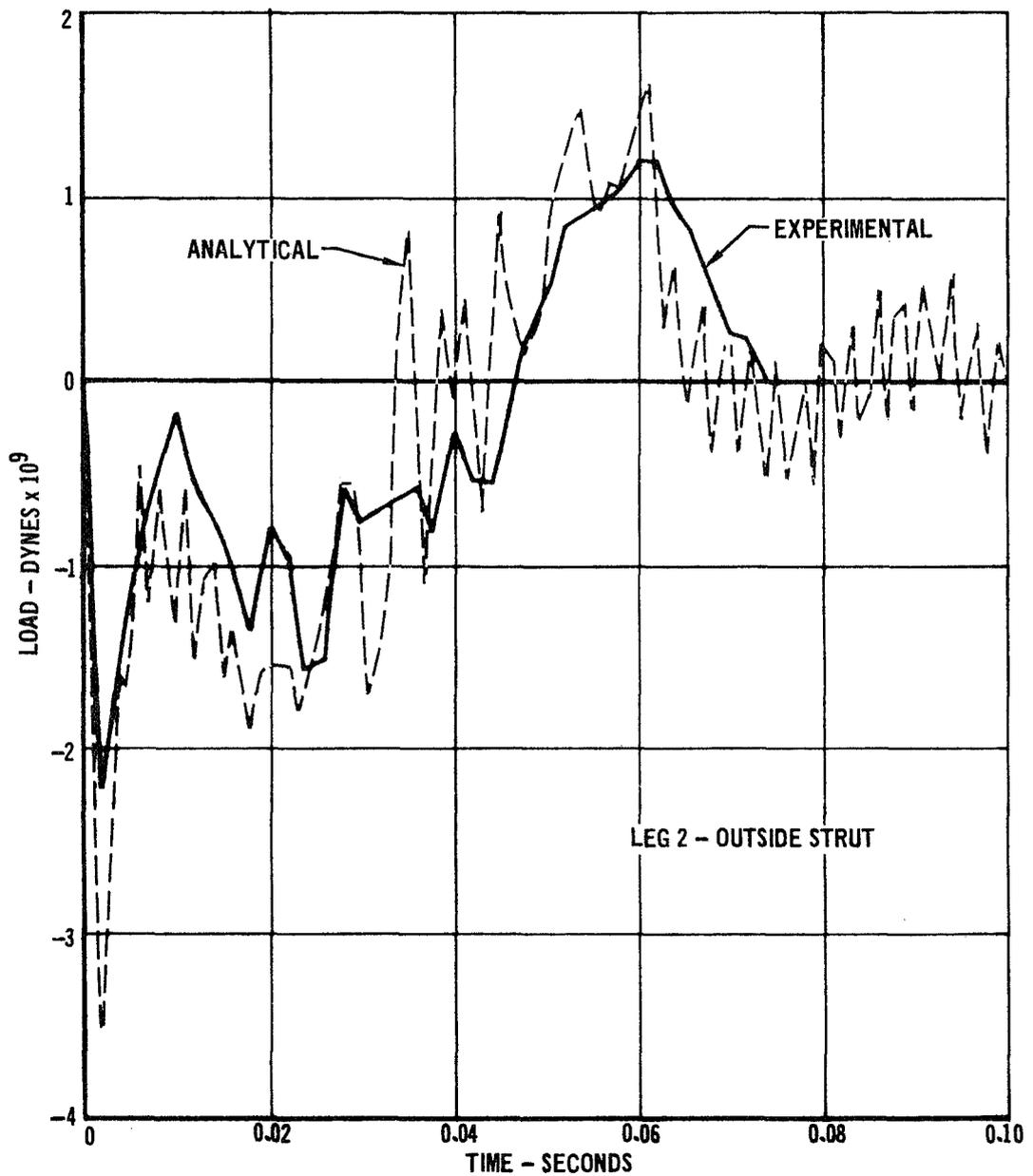


FIGURE 6-20 OUTSIDE DRAG STRUT LOAD WITH FLEXIBLE CENTER BODY

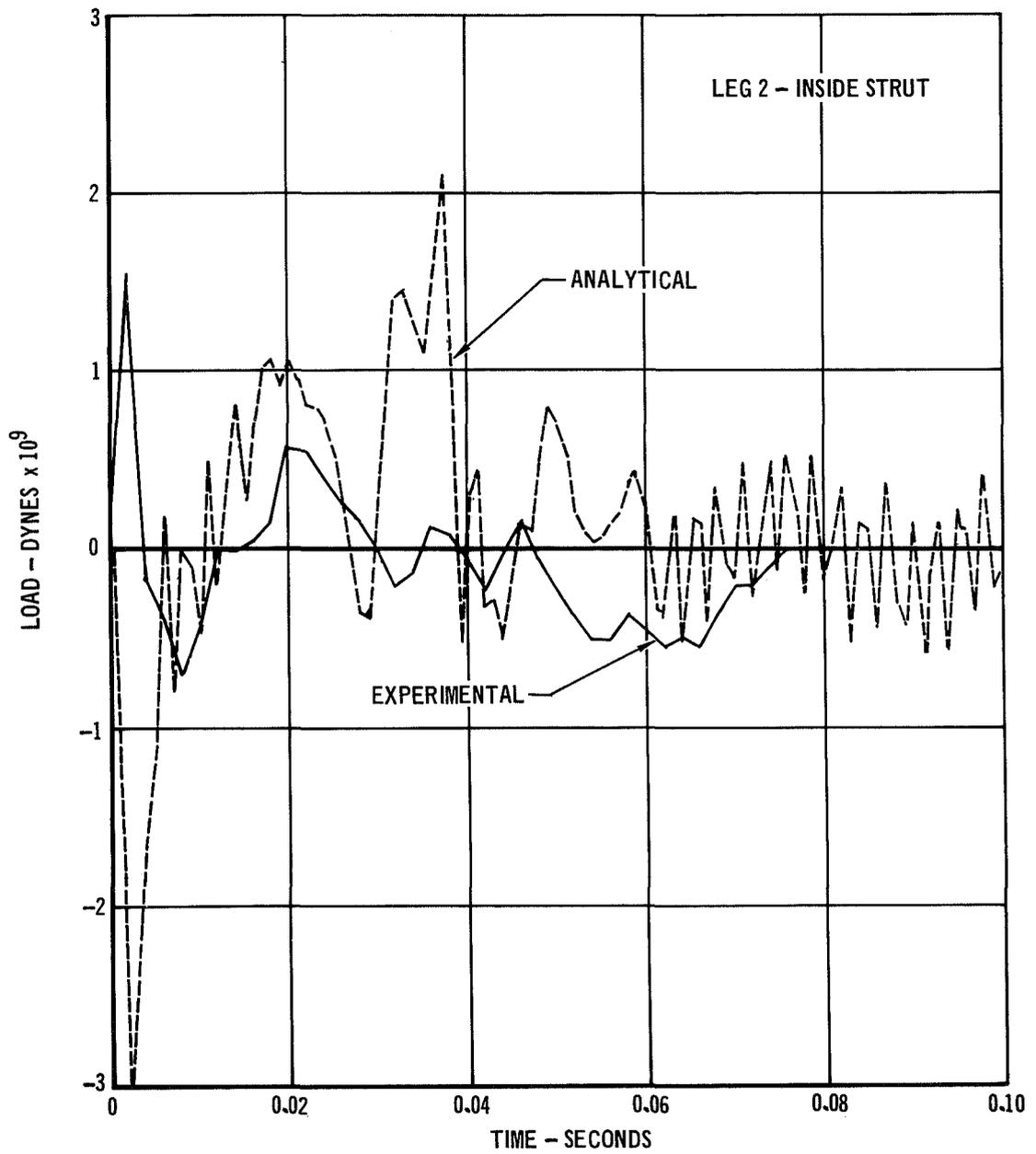


FIGURE 6-21 INSIDE DRAG STRUT LOAD WITH
 FLEXIBLE CENTER BODY

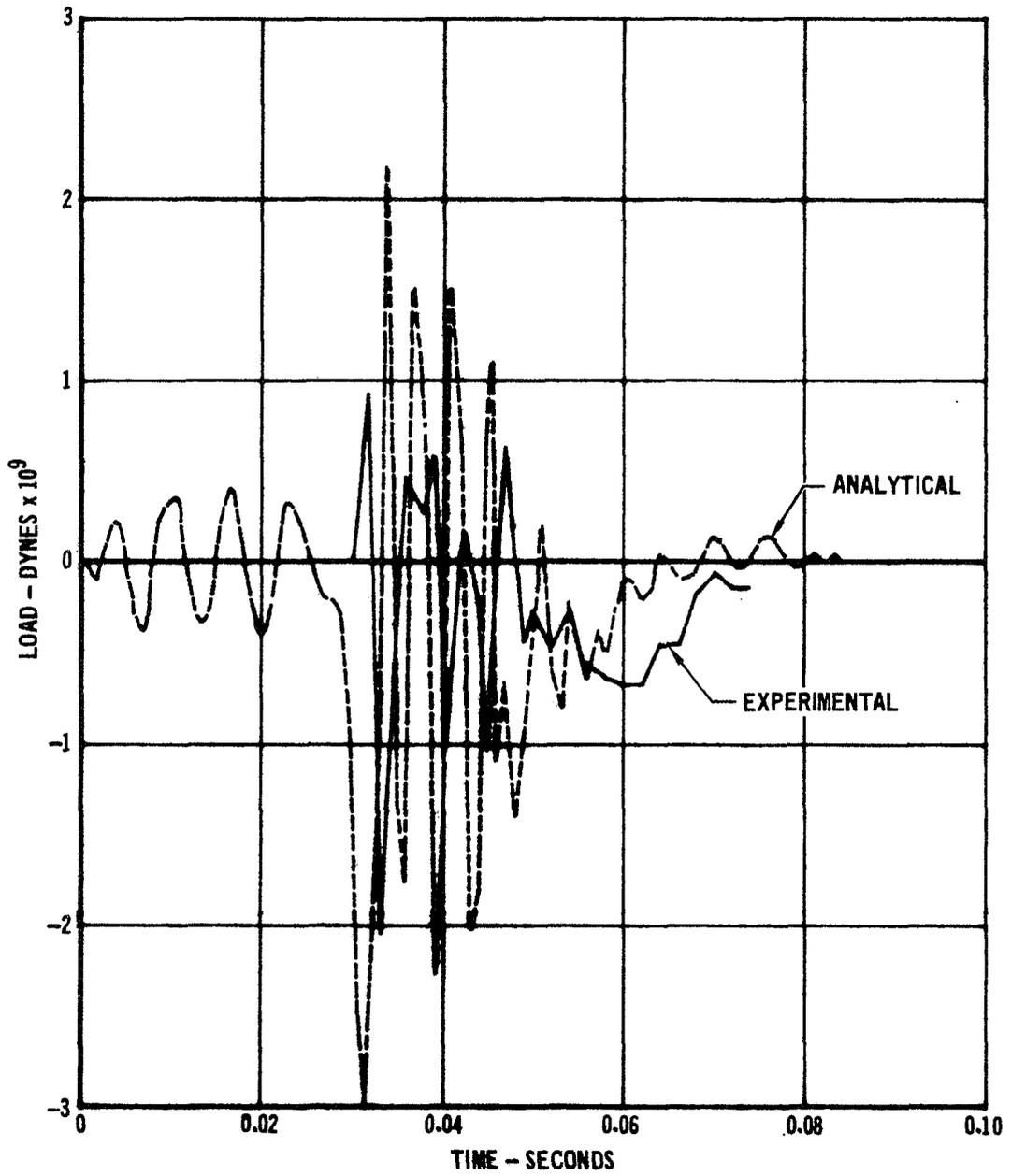


FIGURE 6-22 LEG 1 DRAG STRUT LOAD WITH FLEXIBLE CENTER BODY

6.2.3 Center Body Landing Loads - Observing the center of gravity acceleration time histories shown in Figure 6-17, it is seen that there are several points of apparent high acceleration of the center body structure. The time history data stored on magnetic tape during this run was retrieved with the Center Body Landing Loads Program and total load distributions on the center body were determined at several of these points during the landing. The load distribution at the landing time of 0.011 seconds was selected as critical because it yielded the largest loads in the -Z direction at the center body center of gravity. Appendix E presents the load distribution obtained with the Center Body Landing Loads Program for this time.

6.2.4 Center Body Internal Load Distribution - The Center Body Option of the Structural Analysis Program was employed to determine joint displacements and internal load distributions in the center body structure for landing loads defined in Section 6.2.3 at a time of 0.011 seconds. The center body idealization shown in Figure 6-8 was employed. A listing of input information and output data for this problem is presented in Appendix A. Header card (see Figure 4-32) for this data case does not appear in the output listing although it was the first card of the data set.

In the center body finite element idealization, attach points for legs 2 and 3 were assumed to be joints 12, 15, 17, 22, 25, and 27. This is an approximation since the physical attach points for the drag struts are located at the ends of clevis fittings which are adjacent to the above joints. The center body was supported at joints 12, 15, 17, 22, 25 and 27 which were assumed to be pinned with zero translational displacements and zero applied moments. The center body landing loads distribution at 0.011 seconds is given in Appendix E. Due to the assumed drag strut locations in the center body idealization, unbalances in these loads are indicated, especially moment about the lander Y axis. To insure a static balance, the negative of these unbalance loads were applied at the center of gravity (joint 33) in addition to the three force components at this joint. For computer input, nominal values were selected by default for all data option indicators except INDITR and TMAX. INDITR (maximum number of iterations allowed) was set at 2000. TMAX (maximum CP seconds allowed for solution) was specified as 100.

Included in computer output are displacements and external loads for

all joints and internal loads in all bar elements. The displacements at the center of gravity and the internal bending moments about the local Z axis for the radial beam (elements 46 through 51) are of particular significance. Results of this analysis are compared with the results for a redesigned center body structure in Figure 6-28, Section 6.3.

In order to emphasize the importance of proper idealization of the structure when determining displacements and internal loads, the loading condition defined above was applied to a refined idealization of the Task Order Three Lander Center Body as discussed in Appendix G. Displacements and internal loads obtained when employing the refined idealization are different than those obtained when employing the idealization in Figure 6-8 and probably are more accurate. For the purpose of demonstrating program capabilities, the idealization shown in Figure 6-8 is entirely adequate and will be employed in the following sections.

It must be kept in mind that a more refined idealization requires more computer run time. For example, run time for the idealization in Figure 6-8 was 46 CP seconds while that for the refined idealization in Appendix G was 106 CP seconds. A compromise between idealization refinement and computer run time must be made.

6.3 Analysis of Modified Task Order Three Lander - The analysis of the Task Order Three lander, Section 6.2, indicated considerable flexibility of the center body structure in the lander Y-Z plane. This inplane flexibility is indicated in the measured Z center of gravity acceleration, Figure 6-15, obtained during the NASA drop test program.

A modification of the Task Order Three lander center body structure to increase the stiffness and improve the load carrying capability in the Y-Z plane was studied. Three configurations, incorporating the addition of a number of tension straps, were considered in arriving at the Modified Task Order Three lander. These consisted of 9-strap, 12-strap, and 18-strap configurations. The 12-strap configuration is shown in Figure 6-23. The 9-strap configuration was the same as the 12-strap configuration with the upper straps 63, 64, and 65 removed. The 18-strap configuration was like the 12-strap with six additional straps added to the upper portion of the center body. These upper straps were located in a manner similar to the lower straps 57 through 62.

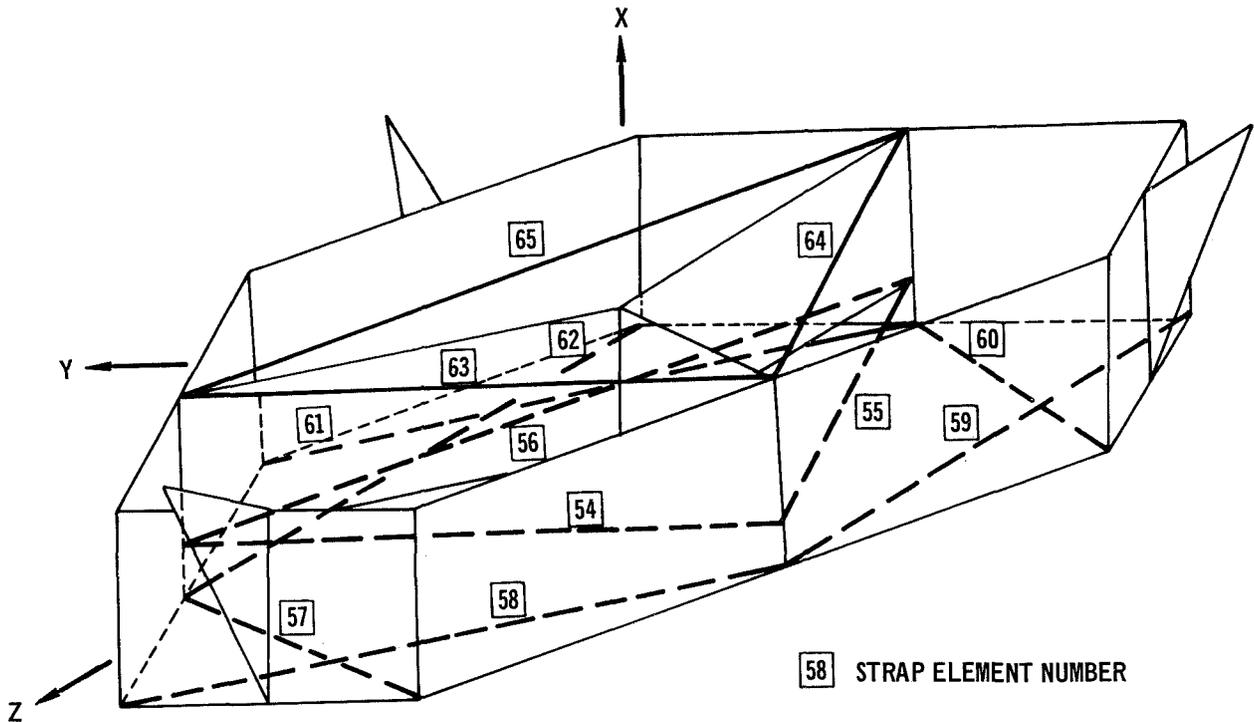
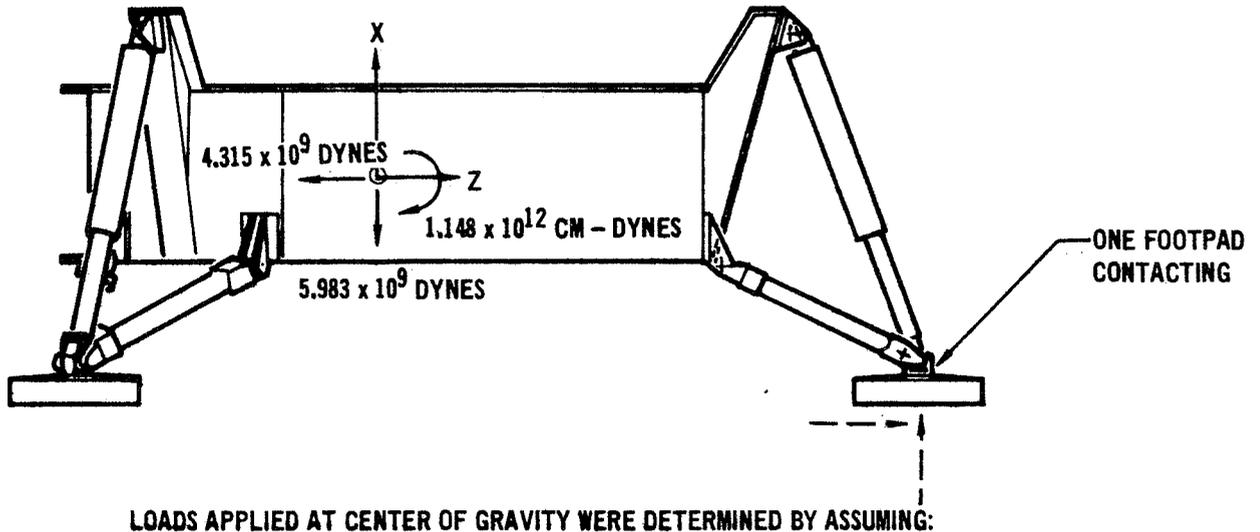


FIGURE 6-23 CENTER BODY IDEALIZATION INCLUDING TENSION STRAPS

A representative loading condition was applied to a finite element idealization of these three strap configurations employing the Center Body Option of the Structural Analysis Program. For this study, the basic center body structure was idealized as shown in Figure 6-8. Each set of tension straps was then added to this idealization, as indicated in Figure 6-23 for the 12-strap configuration. In each case, all straps were assumed to have a cross-sectional area of 1.29 square centimeters and a modulus of elasticity of 2.068×10^{12} dynes/cm².

Figure 6-24 shows the loading condition which was employed to evaluate each of the tension strap configurations. This condition was found to be the critical one when the most severe center of gravity accelerations noted during the analysis of the Task Order Three lander in Section 6.2 were combined for one and two leg landings. Employing the criterion of reducing both the radial beam bending moments about the local Z axes and inplane displacements,

the comparison shown in Figure 6-25 was made for the three tension strap configurations. Based on the results shown in this figure, the 12-strap configuration was chosen as the Modified Task Order Three lander. This configuration resulted in a 3.1 reduction factor for radial beam bending moment and a 5.45 multiplication factor for center body inplane stiffness when compared to the Task Order Three lander structure. The 18-strap configuration was not selected because the slight improvement in the parameters of interest was not sufficient to justify six additional straps. The weight of the twelve tension straps was removed from the ballast weights in the idealization of the modified Task Order Three configuration to maintain the same total lander weight.



LOADS APPLIED AT CENTER OF GRAVITY WERE DETERMINED BY ASSUMING:
 + X ACCELERATION OF 50 g's
 + Z ACCELERATION OF 36 g's
 WEIGHT OF HUB AND RADIAL BEAMS = 1.196×10^8 DYNES,
 CONCENTRATED AT CENTER OF GRAVITY

THEREFORE, LOADS SHOWN ARE:

$$F_X = -m\ddot{X} = -5.983 \times 10^9 \text{ DYNES}$$

$$F_Z = -m\ddot{Z} = -4.315 \times 10^9 \text{ DYNES}$$

$$M_Y = \text{MOMENTS SUMMED ABOUT CONTACTING FOOTPAD} = -1.148 \times 10^{12} \text{ CM-DYNES}$$

FIGURE 6-24 LOADING CONDITION FOR EVALUATION OF MODIFIED TASK ORDER THREE LANDER CENTER BODY STRUCTURE

STRAP CONFIGURATION	MAXIMUM RADIAL BEAM TRANSVERSE BENDING MOMENT* (CM-DYNES ULT.)	CENTER BODY INPLANE STIFFNESS** (DYNES/CM)
NONE	8.203×10^{10}	1.601×10^9
9	3.378×10^{10}	5.366×10^9
12	2.649×10^{10}	8.748×10^9
18	1.625×10^{10}	9.159×10^9

* BENDING MOMENTS ARE ABOUT LOCAL Z AXIS OF ELEMENT AT "Q" END

** CENTER BODY INPLANE STIFFNESS WAS OBTAINED BY APPLYING A UNIT LOAD AT THE CENTER OF GRAVITY IN THE -Z GLOBAL DIRECTION. THE STIFFNESS FACTOR IS THIS LOAD DIVIDED BY THE RESULTING CENTER OF GRAVITY GLOBAL Z DISPLACEMENT

FIGURE 6-25 COMPARISON OF VARIOUS TENSION STRAP CONFIGURATIONS

Employing the finite element idealization shown in Figures 6-8 and 6-23, a modal analysis was performed on the Modified Task Order Three lander center body. Three modes, with frequencies of 75.2 Hz, 149 Hz, and 163 Hz, were obtained whose modal deformations were predominantly in the lander Y-Z plane. These modal frequencies are much higher than the comparable frequencies obtained for the Task Order Three Lander, Section 6.2.1, thus illustrating the stiffening effect of the straps.

The drop test three landing conditions were used to evaluate the Modified Task Order Three lander center body structure. Using the above three modes to represent the center body flexibility effects, the accelerations and drag strut load shown in Figures 6-26 and 6-27 were predicted with the Landing Loads and Motions Program. These quantities are superimposed on the predicted results obtained for the original Task Order Three lander center body. The oscillatory nature of the Z acceleration is due to the large excitation of the 75.2 Hz mode for the initial conditions considered.

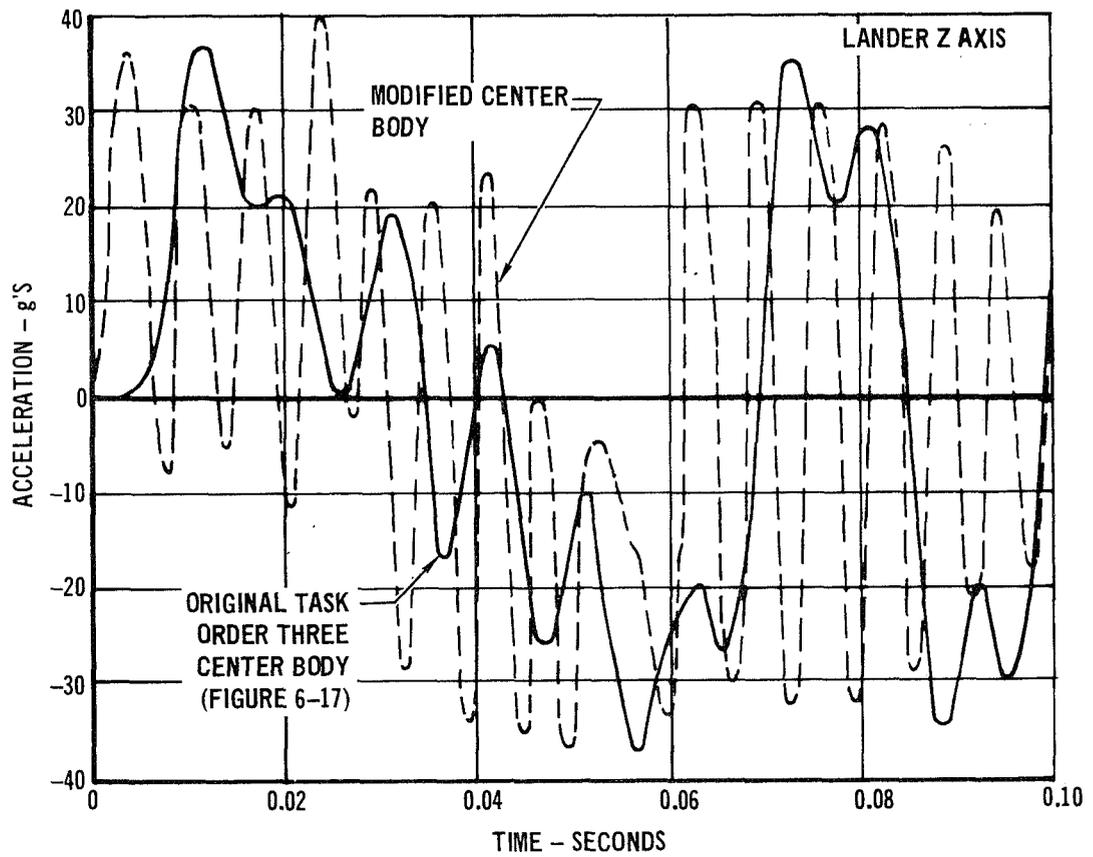
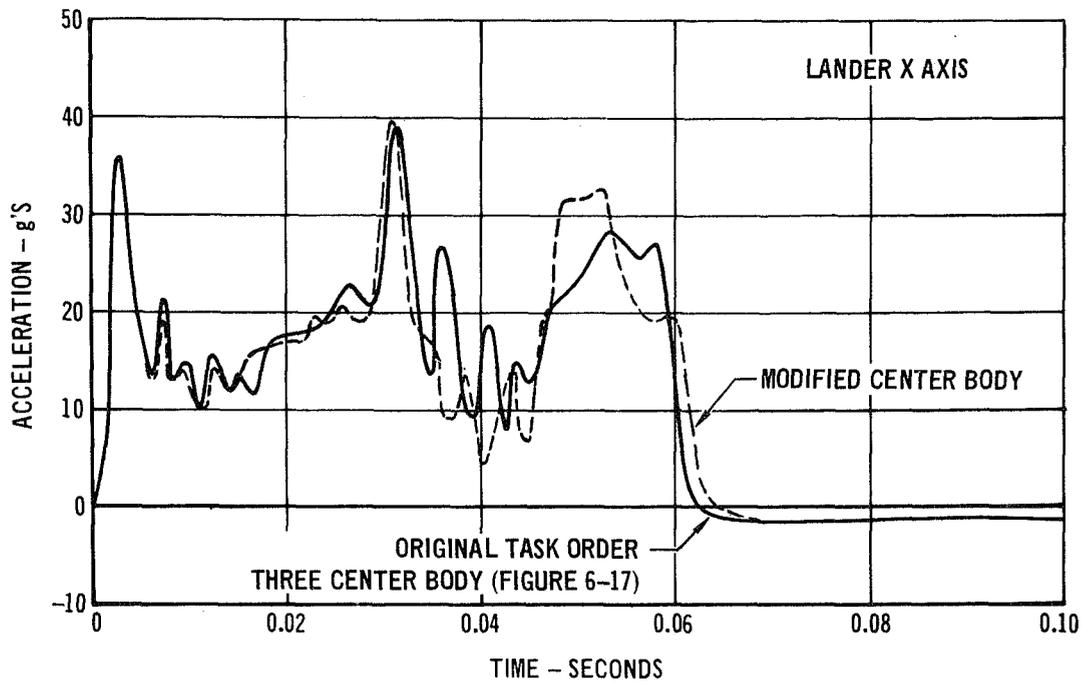


FIGURE 6-26 CENTER OF GRAVITY ACCELERATIONS WITH MODIFIED CENTER BODY STRUCTURE

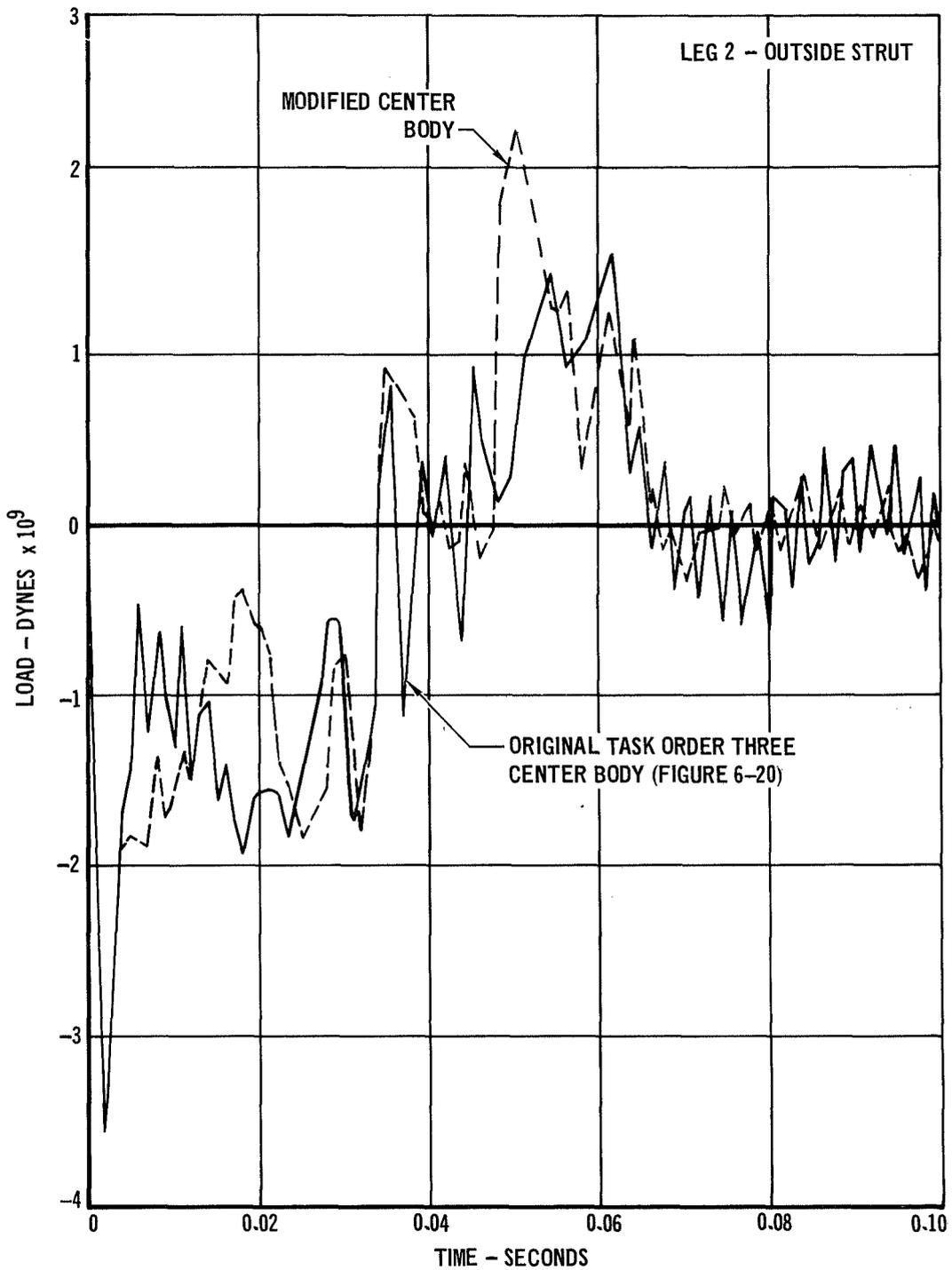


FIGURE 6-27 OUTSIDE DRAG STRUT LOAD WITH MODIFIED CENTER BODY STRUCTURE

The distribution of landing loads throughout the center body structure was obtained for a number of points during the landing time history. At a real time of 0.011 seconds, the center body experienced high loads and this point was chosen for analysis with the Center Body Option of the Structural Analysis Program. Employing the center body finite element idealization shown in Figures 6-8 and 6-23, internal member loads and joint displacements at this time were determined. A comparison of these loads and displacements with the loads and displacements obtained for the Task Order Three lander, Section 6.2.4, are presented in Figure 6-28. As can be seen, the radial beam bending moments in the Modified Task Order Three center body are lower than those obtained for the Task Order Three Center body. In addition, the global Z displacement of the center body center of gravity is less for the Modified Task Order Three center body. This trend of lower member loads and reduced displacements is a result of the additional center body load paths and increased stiffness provided by the tension straps.

6.4 Landing Analysis of Full-Mass Task Order Three Lander - Several additional studies for the Task Order Three lander were performed. For all of the following discussions, the structural configuration, landing gear properties, and footpad idealization were the same as discussed in Section 6.2. The mass properties were that of a full mass Task Order Three lander and a local acceleration of gravity of 375 cm/sec^2 was employed.

A lander, horizontal with respect to the local acceleration of gravity, with a vertical velocity of -610 cm/sec along the gravity vector, and landing on a 10° slope, was considered. The initial orientation of the lander was such that a single footpad impacted the landing surface. For comparison, three short runs were made with these initial conditions and the following soil properties.

SS Dense Soil (Figure 5-10):

Primary soil mechanics routine

Relative soil density = .69

Soil unit weight = 1681 dynes/cm^3

Soil internal friction angle = 39°

SS Loose Soil (Figure 5-10):

Primary soil mechanics routine

Relative soil density = 0

Soil unit weight = 1488 dynes/cm^3

Soil internal friction angle = 29°

Secondary Soil:

Secondary soil mechanics routine

Soil elasticity constant = $144 \times 10^7 \text{ dynes/cm}^3$

Coefficient of friction = 0.9

The Secondary Soil is the same soil idealization as that employed during the correlation studies conducted for the third drop of the Task Order Three lander discussed in Section 6.2.2.

	JOINT 33 DISPLACEMENT (CM)		
	X	Y	Z
TASK ORDER THREE LANDER (SECTION 6.2.4)	-9.577×10^{-2}	-1.322×10^{-4}	-4.289×10^{-1}
MODIFIED TASK ORDER THREE LANDER	-4.794×10^{-2}	-1.555×10^{-4}	-7.834×10^{-2}

	RADIAL BEAM BENDING MOMENT ABOUT LOCAL Z AXIS AT "Q" END (CM - DYNE)					
	ELEMENT 46	ELEMENT 47	ELEMENT 48	ELEMENT 49	ELEMENT 50	ELEMENT 51
TASK ORDER THREE LANDER (SECTION 6.2.4)	8.652×10^9	8.022×10^9	-3.648×10^9	-8.028×10^9	5.625×10^5	-3.108×10^6
MODIFIED TASK ORDER THREE LANDER	1.863×10^9	1.573×10^9	-1.869×10^9	-1.611×10^9	1.818×10^7	3.716×10^6

FIGURE 6-28 COMPARISON OF COMPUTER RESULTS FOR ORIGINAL AND MODIFIED TASK ORDER THREE LANDER

Figure 6-29 presents a comparison between the resulting main strut time histories for the three soils considered. As can be seen, the largest main strut load and stroking motion resulted when the Secondary Soil was used. This is reasonable since these soil properties were used to idealize the very rigid landing surface of the Task Order Three lander drop test program. The main strut stroke was much less for the SS Loose Soil than for the other soils.

A second set of initial conditions for the full mass lander were as follows:

Ground slope = 10°

Initial rotation about X_g axis = 20°

Vertical velocity = -610 cm/sec

For this run, the SS Dense Soil employed in the previous case was used.

Figures 6-30 and 6-31 present the accelerations and main strut loads resulting from this spatial landing. The spatial nature of this landing is obvious from these time histories. The main struts in each leg load up sequentially as their respective footpads impact the landing surface. These loads then drop off as the lander's initial kinetic energy is absorbed by the crushable struts.

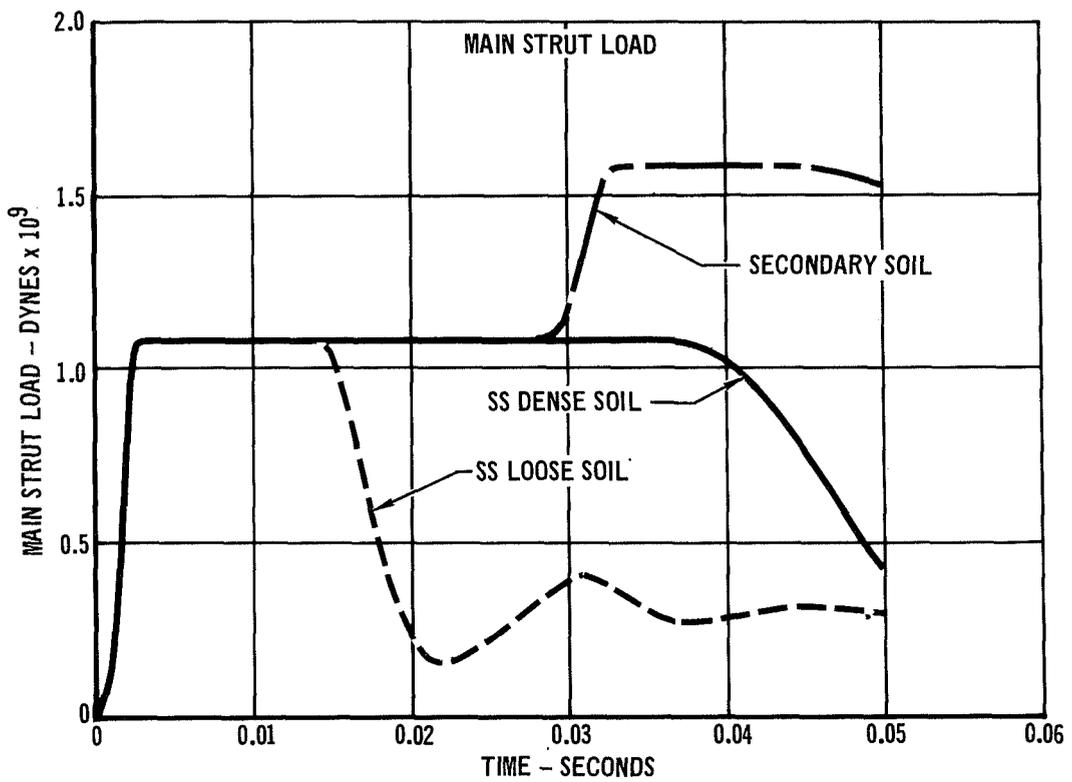
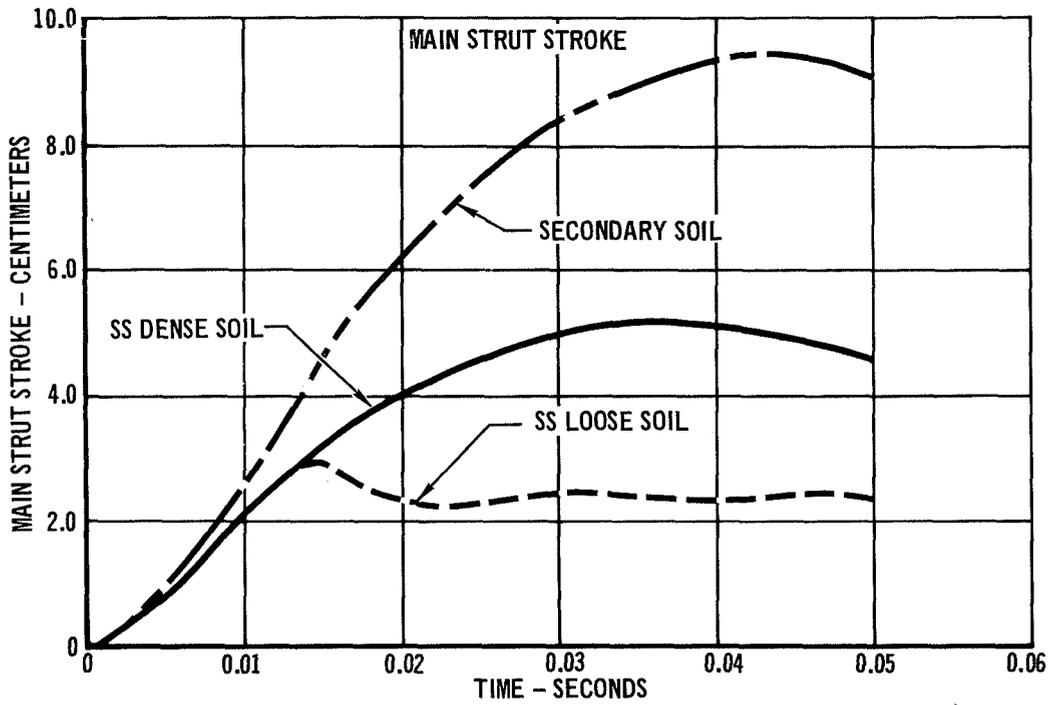


FIGURE 6-29 MAIN STRUT TIME HISTORIES FOR VARIOUS LANDING SURFACE SOIL IDEALIZATIONS

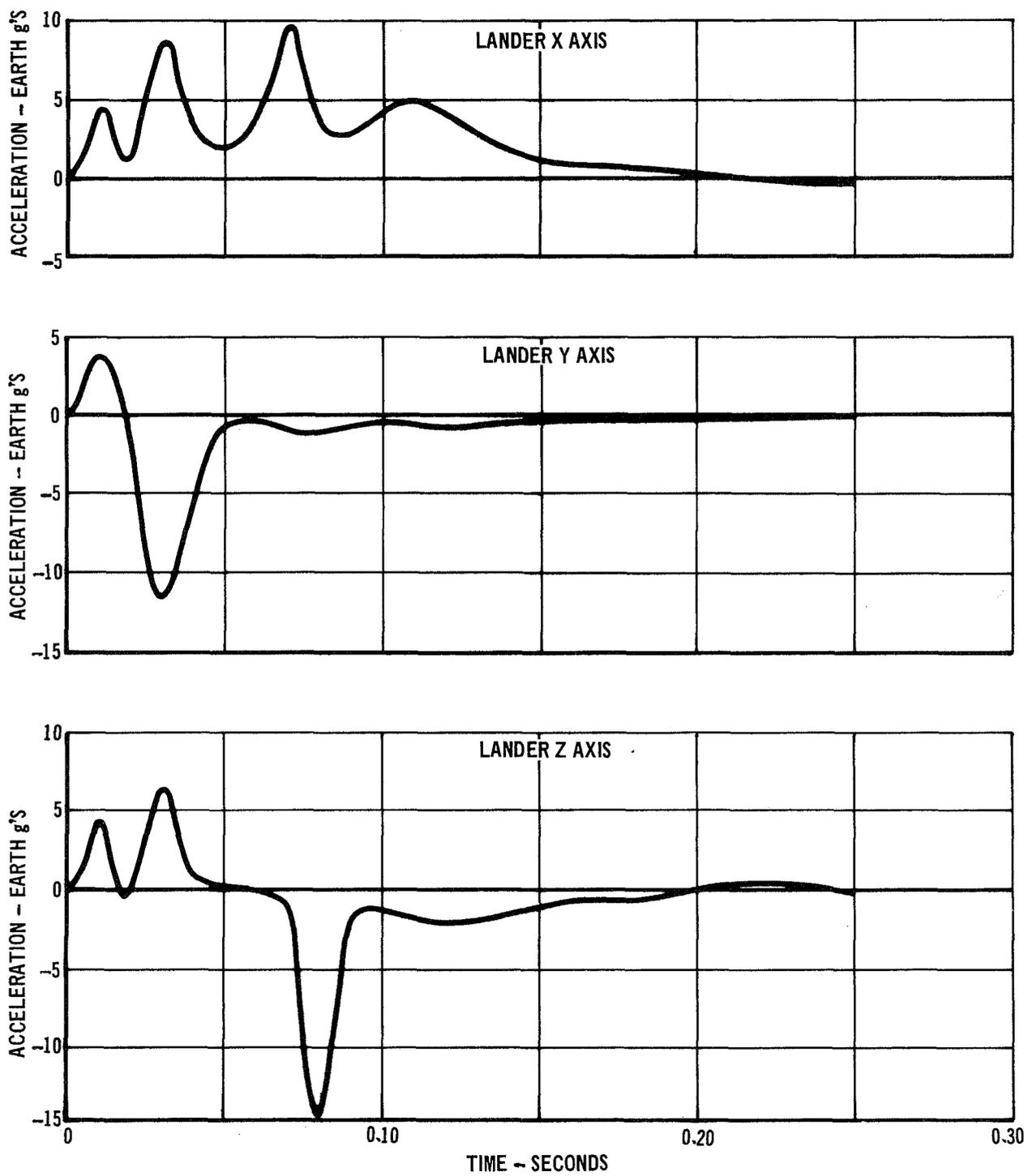


FIGURE 6-30 FULL-MASS TASK ORDER THREE LANDER CENTER OF GRAVITY ACCELERATION TIME HISTORIES

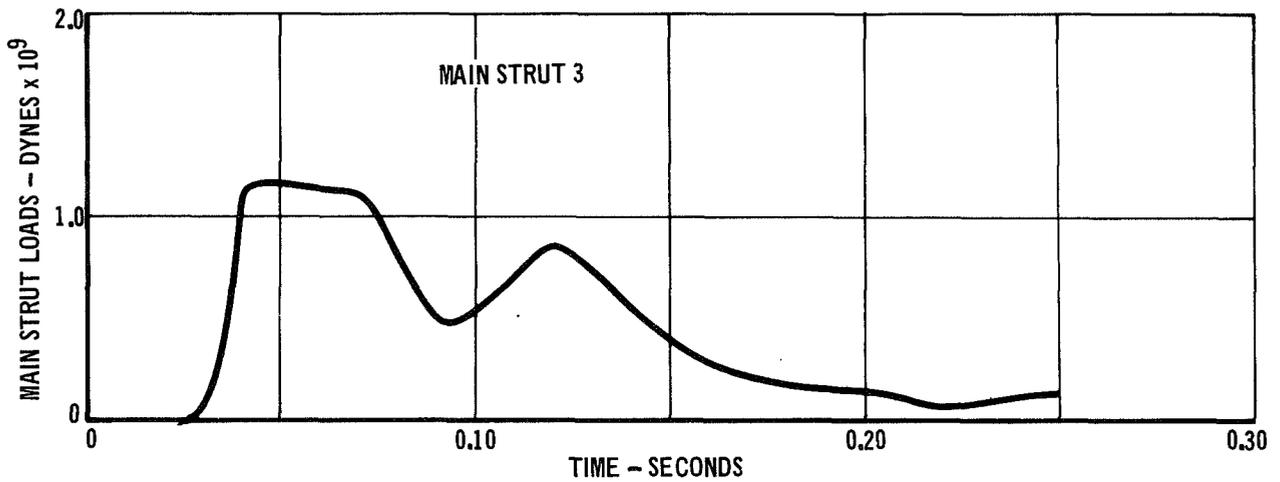
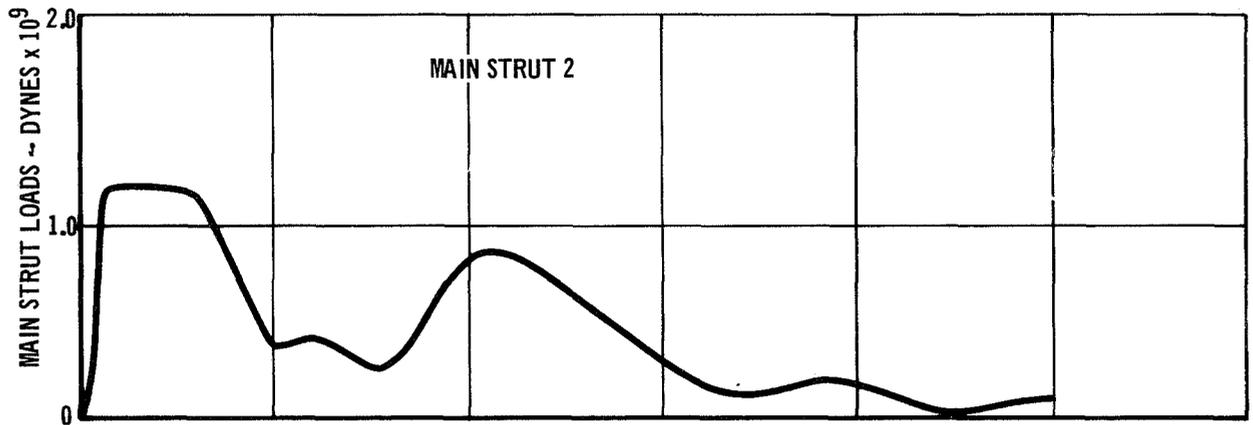
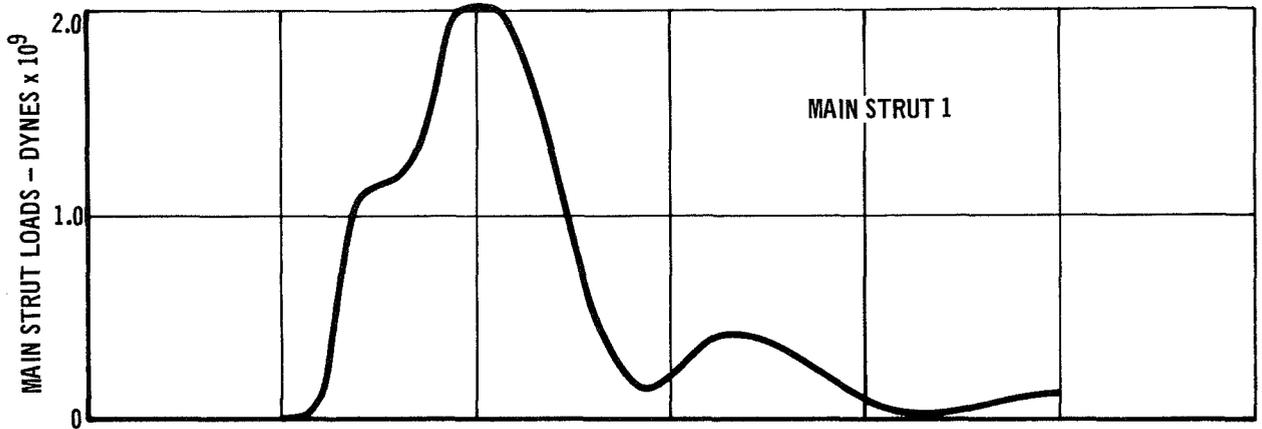


FIGURE 6-31 FULL-MASS TASK ORDER THREE LANDER MAIN STRUT LOAD TIME HISTORIES

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7. CONCLUSIONS

Two methods of analysis and associated computer programs were developed for the investigation of legged landers. One program, the Landing Loads and Motions Program, is used to predict landing loads and spatial motions of a lander. The second, the Structural Analysis Program, determines the internal load distributions in the lander center body structure or frequencies and mode shapes for a free-free center body structure. Energy absorption characteristics of an individual landing gear are also determined with this program. Each program contains options and capabilities which were shown to be important in the analysis of legged lander configurations. For example, in the Landing Loads and Motions Program, options to include the effect of center body flexibility and a comprehensive soil mechanics routine were included. An option to investigate large displacement stroking behavior of a landing gear was incorporated in the Structural Analysis Program.

In addition to the limited analyses made within the constraints of Task Order Five, comparisons of experimental and analytical results for ten additional unpublished tests conducted at the NASA Langley Research Center show that the programs are working satisfactorily. However, the effects of friction between the footpads and the landing surface are very difficult to simulate analytically because of the uncertainties involved in defining these effects during dynamic model testing. The limited analyses presented in this report do serve to demonstrate primary program capabilities. As a result of these analyses, the following conclusions are possible:

- (1) Center body flexibility significantly affects landing loads and motions. This was illustrated during the landing studies conducted on the Task Order Three lander, Section 6.2.2. For a rigid center body, the drag strut loads (Figure 6-16) were higher than with a flexible center body (Figure 6-20). In addition, the center of gravity accelerations in the rigid center body (Figure 6-15) built up more rapidly than was indicated in either the test data or in the analysis with flexibility included (Figure 6-17). Very good correlation with test data resulted when center body flexibility was included.

- (2) Soil properties significantly affect landing gear strut behavior. In Figure 6-29, it was shown that much higher landing gear strut strokes and loads result when landing on hard soil. When landing in loose sand, strut stroke increased gradually with time with resulting lower strut loads. Overall lander motions are highly dependent upon soil properties. Therefore, it is important that programs for studying landing motions, include the capability for properly representing soil characteristics.
- (3) Care must be exercised in the finite element idealization of the center body structure. As discussed in Section 6.2, the predicted internal load distributions were different for two different idealizations of the same structure. However, it was also shown that the first idealization presented in Section 6.2 resulted in very good center body flexibility data for use with the Landing Loads and Motions Program. In general, a more sophisticated idealization is required for determination of internal loads than for generation of center body flexibility information for the landing program.
- (4) Landing gear orientation and landing surface friction greatly influence the large displacement stroking behavior of a landing gear configuration. Comparison between Figures 6-2 and 6-3 indicate that the drag strut loads in an inverted tripod gear are much different for two different gear orientations. As shown in Figure 6-6, the energy absorption capabilities of a landing gear are dependent on the coefficient of friction employed and the orientation of the gear with respect to the landing surface.
- (5) Increasing center body stiffness does not necessarily reduce peak center of gravity accelerations. As indicated in Figure 6-26, the peak predicted accelerations for the original Task Order Three center body and the Modified Task Order Three center body were of the same magnitude. The increased stiffness of the modified structure resulted in a higher frequency oscillation of the Z acceleration. However, in this case, the individual member loads were reduced due to the additional load paths available in the structure.

8. REFERENCES

1. Otto, O. R., Melliore, R. A., and Lopatin, A.: "Design and Fabrication of a Full-Size Landing Impact Test Model of the Mars Legged Lander Configuration," NASA CR-111771, September 1970.
2. Otto, O. R., Dorr, D. J., Laurenson, R. M., Burton, D. J., and Moore, R. L.: "Analyses and Limited Evaluations of Payload and Landing System Structures for the Survivable Soft Landing of Instrument Payloads," NASA CR-66914, April 1970.
3. Serpanos, J. E.: "A User-Coded Matrix Generator for the Displacement Method - Format II Second Version of Fortran Matrix Abstraction Technique," Air Force Flight Dynamics Laboratory Technical Report 66-207, Volume IV, March 1967.
4. Varga, R. S.: Matrix Iterative Analysis, Prentice Hall, Inc., Englewood Cliffs, N. J., 1962.
5. Hestenes, M. and Stiefel, E.: "Method of Conjugate Gradients for Solving Linear Systems," Report 1659, Nation Bureau of Standards, 1952.
6. Hurty, Walter C. and Robinstein, Moshe F.: Dynamics of Structures, Prentice-Hall, INC., Englewood Cliffs, N. J., 1964, pp 43-45.
7. Young, Ronald and McCallum, Helen: "The Determination of Frequencies and Modes of Undamped Structures Using Finite-Element Stiffness and Consistent-Mass Matrices," Aeroelastic and Structures Research Laboratory, Massachusetts Institute of Technology, Cambridge, Massachusetts, June 1965.
8. Martin, H. C.: "On the Derivation of Stiffness Matrices for the Analysis of Large Deflection and Stability Problems," Proceedings of the Conference on Matrix Methods in Structural Mechanics, TR-66-80, 1966, Wright-Patterson AFB, Ohio.
9. Thomson, William T.: Vibration Theory and Applications, Prentice-Hall Inc., Englewood Cliffs, N. J., 1965, pp 294-299.
10. Bisplinghoff, Raymond L., Ashley, Holt, and Lafman, Robert L.: Aeroelasticity, Addison-Wesley Publishing Company, Inc., Cambridge, Massachusetts, 1955, pp 632-635.

11. "Summary Report - Lunar Module (LM) Soil Mechanics Study," Report #AM-68-4, Bendix Energy Controls Division, South Bend, Indiana, November 1968.
12. Anderson, R. G.: "Application of LM Soil-Footpad Interaction Model to Surveyor Landing Dynamics Simulation," Paper 68K11, Bendix Energy Controls Division, South Bend, Indiana, November 1968.

APPENDIX A

EXAMPLE OF INPUT AND OUTPUT

CENTER BODY OPTION

STRUCTURAL ANALYSIS PROGRAM

STRUCTURAL ANALYSIS PROGRAM ---- LEGGED LANDER
 MASTER AGREEMENT, CONTRACT NAS1-B137, TASK ORDER NUMBER FIVE
 MC DONNELL DOUGLAS AERONAUTICS COMPANY, EAST

STRUCTURAL ANALYSIS DATA - CARD CODE

PLANK - 0
 1
 2
 3
 4
 5
 6
 7
 8

COMMENTS
 NODAL POINT DEFINITIONS
 REFERENCE POINTS
 NODAL POINT RESTRAINT DEFINITIONS
 FORCE VECTORS
 MOMENT VECTORS
 BAR DEFINITIONS
 SHEAR PANEL DEFINITIONS
 FORMATED-DATA TERMINATOR

PLANK	0	1	2	3	4	5	6	7	8
1	11.5833	31.7245	85.7758						
1	-28.6573	31.7246	85.7758						
1	13.5833	0.	85.7758						
1	-28.6670	0.	85.7758						
1	23.4834	0.	115.538						
1	13.5833	-31.7245	85.7758						
1	-28.6673	-31.7246	85.7758						
1	13.5830	-60.9345	35.1790						
1	-17.9074	-60.9345	35.1790						
1	-28.6673	-60.9345	35.1790						
1	13.5833	-90.1445	-15.4179						
1	-28.6673	-90.1445	-15.4179						
1	13.5830	-74.295	-42.8752						
1	-28.6673	-74.295	-42.8752						
1	29.4834	-98.3234	-56.769						
1	13.5830	-58.42	-70.358						
1	-28.6673	-58.42	-70.358						
1	13.5833	0.	-70.358						
1	-28.6670	0.	-70.358						
1	13.583	58.42	-70.358						
1	-28.667	58.42	-70.358						
1	13.583	74.295	-42.8752						
1	-28.667	74.295	-42.8752						
1	29.4834	98.3234	-56.769						
1	13.583	90.1446	-15.4179						
1	-28.667	90.1446	-15.4179						
1	13.583	50.9346	35.179						
1	-17.9073	50.9346	35.179						
1	-28.667	50.9346	35.179						
1	13.583	0.	0.						
1	-17.907	0.	0.						
1	0.	0.	0.						

JOINT INFORMATION CARDS
 FORCE AND MORE IT LIMIT CARDS
 SPECIFIED FORCE VECTOR CARDS

4 5 4.91E+06 2.310E+03 5.492E+07
 4 1.873E+07 4.510E+03 4.372E+07
 4 7-1.086E+08 1.265E+04 3.742E+08
 4 8-2.922E+06 7.703E+07 2.309E+06
 4 9-1.875E+06 3.736E+07 7.265E+05
 4 10-1.839E+06 3.220E+07 1.003E+07
 4 11-2.505E+07 3.670E+07 2.372E+07
 4 12-5.855E+07 2.191E+07 1.301E+07
 4 13-3.531E+07 1.743E+05 3.324E+07
 4 14-1.001E+08-1.351E+05 1.254E+06
 4 15-1.17E+08-1.153E+05-1.479E+08
 4 16-6.208E+07 3.229E+07-7.272E+07
 4 17-4.657E+07 4.866E+04-1.652E+07
 4 18-9.941E+07 1.123E+06 1.631E+05
 4 19-5.825E+07-2.199E+07 1.528E+07
 4 20-3.516E+07-1.716E+06 3.307E+07
 4 21-2.518E+07-3.60E+07 2.393E+07
 4 22-3.313E+06-7.598E+07-2.222E+06
 4 23-2.086E+06-3.740E+07 7.221E+06
 4 24-2.033E+06-3.224E+07 9.952E+06
 4 25-7.45E+08 5.454E+05-1.371E+09
 4 26-3.275E+08-3.546E+05-1.04E+09
 4 27-3.560E+07 3.004E+04-1.29E+08

SPECIFIED MOMENT VECTOR CARDS

5 1-9.18E+05 2.357E+03-2.753E+05
 1-9.18E+05 2.357E+03-2.753E+05

9AR INFORMATION CARDS

6	1	1	2	3	11.7742	106.70	1.5775	6.826E11	2.620E11	
6	2	1	3	28	16.1291	111.43	2.1644	6.826E11	2.620E11	
6	3	2	1	30	24.8322	435.21	3.3299	6.826E11	2.620E11	
6	4	3	4	1	14.1123	113.95	6.4557	6.826E11	2.620E11	
6	5	3	5	1	7.6613	55.53	1.0281	6.826E11	2.620E11	
6	6	4	5	2	4.8387	6.98	1.6243	6.826E11	2.620E11	
6	7	3	5	29	16.1291	111.43	2.1644	6.826E11	2.620E11	
6	8	4	7	31	24.8322	435.21	3.3299	6.826E11	2.620E11	
6	9	6	7	3	11.7742	106.70	1.5775	6.826E11	2.620E11	
6	10	6	8	3	9.2774	29.58	1.2445	6.826E11	2.620E11	
6	11	7	10	4	9.2774	29.58	1.2445	6.826E11	2.620E11	
6	12	6	9	11	20.729	255.28	2.7404	6.826E11	2.620E11	
6	13	9	10	12	16.3742	253.57	2.1977	6.826E11	2.620E11	
6	14	8	11	13	9.2774	29.58	1.2445	6.826E11	2.620E11	
6	15	10	12	14	9.2774	29.58	1.2445	6.826E11	2.620E11	
6	16	11	12	13	11.7742	106.70	1.5775	6.826E11	2.620E11	
6	17	11	13	8	16.1291	111.43	2.1644	6.826E11	2.620E11	
6	18	12	14	10	24.8322	435.21	3.3299	6.826E11	2.620E11	
6	19	13	14	11	14.1123	110.95	4.87	6.4557	6.826E11	2.620E11
6	20	13	15	11	7.6613	55.53	2.22	1.0281	6.826E11	2.620E11
6	21	14	15	12	4.8387	6.98	1.6243	6.826E11	2.620E11	
6	22	17	16	18	16.1291	111.43	2.1644	6.826E11	2.620E11	
6	23	14	17	20	24.8322	435.21	3.3299	6.826E11	2.620E11	
6	24	16	17	14	11.7742	106.70	1.5775	6.826E11	2.620E11	
6	25	16	18	13	9.2774	29.58	1.2445	6.826E11	2.620E11	
6	26	17	20	14	9.2774	29.58	1.2445	6.826E11	2.620E11	
6	27	18	19	16	20.729	255.28	2.7404	6.826E11	2.620E11	
6	28	19	23	17	16.3742	253.57	2.1977	6.826E11	2.620E11	
6	29	18	21	23	9.2774	29.58	1.2445	6.826E11	2.620E11	
6	30	21	22	24	9.2774	29.58	1.2445	6.826E11	2.620E11	
6	31	21	22	24	11.7742	106.70	1.5775	6.826E11	2.620E11	
6	32	21	23	18	16.1291	111.43	2.1644	6.826E11	2.620E11	
6	33	22	24	20	24.8322	435.21	3.3299	6.826E11	2.620E11	
6	34	23	24	22	14.1123	113.95	4.87	6.4557	6.826E11	2.620E11
6	35	23	25	21	7.6613	55.53	1.0281	6.826E11	2.620E11	

6	36	24	25	22	4.8387	6.98	3.69	.6243	6.826E11	2.620E11
6	37	23	26	28	16.1291	111.43	274.90	2.1644	6.826E11	2.620E11
6	38	24	27	30	24.8322	435.21	700.43	3.3299	6.826E11	2.620E11
6	39	26	27	24	14.7742	106.70	168.07	1.5775	6.826E11	2.620E11
6	40	26	23	23	9.2774	29.59	4010.2	1.2445	6.826E11	2.620E11
6	41	27	30	24	9.2774	29.59	4010.2	1.2445	6.826E11	2.620E11
6	42	28	29	1	20.7291	255.28	109.46	2.7804	6.826E11	2.620E11
6	43	29	30	2	16.3742	253.57	27.46	2.1977	6.826E11	2.620E11
6	44	1	24	3	9.2774	29.59	4010.2	1.2445	6.826E11	2.620E11
6	45	2	30	4	9.2774	29.59	4010.2	1.2445	6.826E11	2.620E11
6	46	28	31	8	18.0645	47.28	2004.2	1.0822	6.826E11	2.620E11
6	47	29	32	9	18.0645	47.28	2004.2	1.0822	6.826E11	2.620E11
6	48	8	31	18	18.0645	47.28	2004.2	1.0822	6.826E11	2.620E11
6	49	9	32	19	18.0645	47.28	2004.2	1.0822	6.826E11	2.620E11
6	50	18	31	8	18.0645	47.28	2004.2	1.0822	6.826E11	2.620E11
6	51	19	32	9	18.0645	47.28	2004.2	1.0822	6.826E11	2.620E11
6	52	31	33	3	58.7741	2776.3	2776.3	5551.7	6.826E11	2.620E11
6	53	32	33	4	58.7741	2776.3	2776.3	5551.7	6.826E11	2.620E11

ANALYSIS OF TASK ORDER THREE LANDER CENTERBODY STRUCTURE

----- UNITS -----
 FORCE - DYNES
 LENGTH - CM

 DATA TERMINATOR CARD

STRUCTURAL ANALYSIS CONTROL DATA

GENERAL DATA

INDMKT = 0 * 1 IMPLIES READ ALL MATRICES FROM TAPE
 INDSFC = 1 * 0 IMPLIES WRITE GLOBAL BAR MATRICES
 INDSFL = 1 * 0 IMPLIES WRITE LOCAL BAR MATRICES
 INDMKT = 0 * 1 IMPLIES SAVE ALL MATRICES ON TAPE
 INDMIS = 0 * 1 IMPLIES PRINT TOTAL STIFFNESS MATRIX
 ISFDIM = 12304 * MAX. STORAGE FOR STIFFNESS MATRIX

DISPLACEMENT/ROTATION SOLUTION DATA

ERRTOL = 1.000E-04 * ITERATION SOLUTION TOLERANCE
 IND1SL = 0 * 1 IMPLIES AN INITIAL SOLUTION IN SOLVEC
 INDIR = 2000 * MAX. SOLUTION ITERATION CYCLES
 INDIRL = 3 * 1 IMPLIES CONSIDER PLASTICITY
 INDIRX = 2 * ITERATIVE SOLUTION METHOD
 INDMKT = 0 * 1 IMPLIES SAVE SOLVEC ON TAPE
 MINRST = 6 * MIN. ALLOWABLE RESTRAINTS
 RELAXF = 1.00 * RELAXATION FACTOR
 TMAX = 100.000 * ITERATION CP TERMINATION TIME

GLOBAL COORDINATE SYSTEM

GLOBAL POINT DISPLACEMENTS AND ROTATIONS

GLOBAL POINT NUMBER,	DISPLACEMENT X	DISPLACEMENT Y	DISPLACEMENT Z	ROTATION X	ROTATION Y	ROTATION Z
1	-2.394057E-01	-1.284398E-05	9.954863E-02	-2.290412E-03	-2.493686E-03	-3.99032E-05
2	-2.595587E-01	-1.103752E-03	6.875395E-04	-7.508595E-04	-2.517113E-03	-2.217541E-05
3	-2.994768E-01	2.344819E-04	1.365975E-01	1.365980E-05	-3.005784E-03	2.305041E-05
4	-2.596246E-01	-7.071421E-04	1.291253E-02	-1.330393E-05	-2.949074E-03	2.235757E-05
5	-3.47197E-01	2.594447E-04	1.874016E-01	6.590632E-06	-3.100999E-03	1.417395E-05
6	-2.579515E-01	4.806842E-04	9.875488E-02	2.309811E-03	-2.453771E-03	8.500476E-05
7	-2.582035E-01	-3.037241E-04	1.504430E-03	7.269474E-04	-2.476936E-03	6.763974E-05
8	-1.300134E-01	2.193496E-01	-2.929892E-02	1.953775E-03	-2.134581E-03	7.231446E-04
9	-1.300064E-01	1.997026E-01	-9.144344E-02	4.562935E-04	-1.929153E-03	6.764280E-04
10	-1.299991E-01	1.848225E-01	-1.073636E-01	-1.546399E-04	-1.446096E-03	1.831387E-03
11	1.195158E-03	1.044167E-01	3.423583E-02	-3.897371E-03	-1.287170E-03	2.293985E-03
12	0.	0.	0.	-5.674980E-06	-1.318790E-03	2.192291E-03
13	4.343227E-03	2.070061E-02	-1.512224E-02	-1.524434E-03	6.796597E-05	6.018718E-05
14	3.046273E-03	5.448616E-03	2.885772E-03	1.936540E-04	-1.106379E-04	1.956412E-04
15	0.	0.	0.	-6.927687E-04	4.233830E-04	-1.651343E-04
16	-2.75337E-03	-6.087505E-03	-3.239054E-02	-1.848661E-03	5.487169E-04	-2.619314E-05
17	0.	0.	0.	-7.723632E-04	2.783482E-04	1.18644E-05
18	-2.139731E-03	1.646172E-05	-4.03478E-01	-8.232547E-09	-1.378152E-03	-1.181616E-08
19	-2.121140E-03	4.047092E-06	-4.402838E-01	-2.117497E-08	-1.246994E-03	3.966774E-07
20	-1.739390E-03	1.752866E-06	-4.351520E-01	-1.870346E-08	1.379476E-03	1.526218E-08
21	-2.072126E-03	6.124344E-03	-3.239479E-02	1.848546E-03	5.493697E-04	2.617136E-05
22	0.	0.	0.	7.724276E-04	2.790772E-04	-1.115609E-05
23	4.365225E-03	-2.072861E-02	-1.509775E-02	1.531882E-03	6.648816E-05	-6.310193E-05
24	3.161616E-03	-5.449777E-03	2.886710E-03	-1.936656E-04	-1.126262E-04	-1.901363E-04
25	0.	0.	0.	6.889550E-04	4.229063E-04	1.697941E-04
26	1.188954E-03	-1.048444E-01	3.456437E-02	3.914764E-03	-1.295317E-03	-2.314838E-03
27	0.	0.	0.	5.674996E-06	-1.326562E-03	-2.204177E-03
28	-1.307529E-01	-2.196456E-01	-2.894178E-02	-1.975831E-03	-2.150572E-03	-7.172035E-04
29	-1.307460E-01	-2.001335E-01	-9.156317E-02	-4.437621E-04	-1.944082E-03	-6.726332E-04
30	-1.307380E-01	-1.853135E-01	-1.076610E-01	1.813489E-04	-1.464383E-03	-1.884615E-03
31	-9.576417E-02	-4.788775E-05	-4.131009E-01	1.323384E-06	-1.174504E-03	6.175342E-06
32	-9.575433E-02	-2.432919E-04	-4.499126E-01	1.256653E-06	-1.204229E-03	6.149842E-06
33	-9.576677E-02	-1.322386E-04	-4.288944E-01	1.289626E-06	-1.154116E-03	6.222322E-06

1068 ITERATIONS WERE REQUIRED TO REACH A MAXIMUM RELATIVE DIFFERENCE OF 9.6539364E-05

MODAL POINT FORCES AND MOMENTS
GLOBAL COORDINATE SYSTEM

MODAL POINT NUMBR	FORCE			MOMENT		
	X	Y	Z	X	Y	Z
1	1.30100E+07	-7.22700E+03	4.61200E+07	0.	0.	0.
2	-1.9A000E+08	-1.27300E+08	3.73500E+08	0.	0.	0.
3	1.37000E+07	2.08400E+05	5.55900E+07	0.	0.	0.
4	1.13300E+07	-1.54900E+05	4.40300E+07	0.	0.	0.
5	4.91000E+06	2.31800E+05	5.89200E+07	0.	0.	0.
6	1.97300E+07	4.51000E+05	4.57200E+07	0.	0.	0.
7	-1.86600E+08	1.26500E+08	3.74200E+08	0.	0.	0.
8	-2.32200E+06	7.70300E+07	-2.38900E+06	0.	0.	0.
9	-1.97500E+06	3.73800E+07	7.26500E+06	0.	0.	0.
10	-1.33100E+06	3.62000E+07	1.00300E+07	0.	0.	0.
11	-2.30500E+07	3.67000E+07	2.37200E+07	0.	0.	0.
12	-5.23434E+08	-6.86608E+08	3.508397E+08	0.	0.	0.
13	-5.35000E+07	2.19100E+07	1.50000E+07	0.	0.	0.
14	-3.53100E+07	1.74300E+06	3.32400E+07	0.	0.	0.
15	5.49456E+08	-8.074434E+08	-5.364948E+08	0.	0.	0.
16	-1.00000E+08	-1.35100E+06	1.25100E+06	0.	0.	0.
17	7.32102E+08	3.767355E+08	1.001838E+09	0.	0.	0.
18	-1.17000E+08	-1.15300E+02	-1.47900E+08	0.	0.	0.
19	-6.28800E+07	3.22900E+04	-7.27700E+07	0.	0.	0.
20	-4.55700E+07	4.86600E+04	-4.65200E+07	0.	0.	0.
21	-9.34100E+07	1.12300E+06	1.63300E+06	0.	0.	0.
22	7.308529E+08	-3.776764E+08	1.001466E+09	0.	0.	0.
23	-5.42500E+07	-2.19900E+07	1.52900E+07	0.	0.	0.
24	-3.51600E+07	-1.71600E+06	3.30790E+07	0.	0.	0.
25	5.309640E+08	8.099014E+08	-5.387859E+08	0.	0.	0.
26	-2.51400E+07	-3.66000E+07	2.39300E+07	0.	0.	0.
27	-5.712072E+08	6.851741E+08	3.521723E+08	0.	0.	0.
28	-3.51300E+06	-7.69900E+07	-2.22500E+06	0.	0.	0.
29	-2.38600E+06	-3.74000E+07	7.22100E+06	0.	0.	0.
30	-2.33300E+06	-3.22400E+07	9.95200E+06	0.	0.	0.
31	-3.54000E+08	6.45400E+05	-1.37000E+09	0.	0.	0.
32	-3.27600E+08	-3.64600E+05	-1.00400E+09	0.	0.	0.
33	-3.36000E+07	3.05400E+04	-1.29000E+08	-9.18800E+05	2.95700E+09	-2.75800E+06
TOTAL	6.405044E+03	9.765102E+04	6.298171E+03			

RELATIVE SOLUTION ERRORS IN PERCENT
.00023 .00330 .00021

BAR FORCES AND MOMENTS
LOCAL COORDINATE SYSTEMS

BAR NUMREP	NODAL POINT P	NODAL POINT Q	NODAL POINT R	NODAL POINT S	FORCE			MOMENT			FORCE			MOMENT					
					X	Y	Z	X	Y	Z	X	Y	Z	X	Y	Z			
1	1	2	3		POINT P	-4.382243E+07	-2.879997E+07	-1.068451E+08	1.527523E+07	2.191845E+09	2.191845E+09	-6.312003E+08							
					POINT Q	4.882243E+07	2.879997E+07	1.048451E+08	-1.527523E+07	-2.248243E+09	-2.248243E+09	-5.68715E+08							
2	1	3	28		POINT P	4.583158E+07	-6.558738E+07	-1.38447E+07	9.153372E+06	5.913729E+08	5.913729E+08	-4.444382E+09							
					POINT Q	-8.583158E+07	6.558738E+07	1.38447E+07	-9.153372E+06	-1.027751E+08	-1.027751E+08	6.284553E+09							
3	2	4	30		POINT P	2.119086E+08	1.379957E+07	-3.625722E+06	-1.187904E+07	7.361937E+08	7.361937E+08	-6.67698E+09							
					POINT Q	-2.119086E+08	-1.379957E+07	3.625722E+06	1.187904E+07	-6.211691E+08	-6.211691E+08	7.12578E+09							
4	3	4	1		POINT P	3.418606E+07	-2.090930E+05	-5.503318E+05	-1.094831E+06	6.336732E+06	6.336732E+06	-4.52374E+09							
					POINT Q	3.418606E+07	2.090930E+05	5.503318E+05	1.094831E+06	-1.598789E+07	-1.598789E+07	-4.180206E+09							
5	3	5	1		POINT P	-8.918537E+07	-1.328559E+05	-4.698598E+05	9.459827E+04	1.202607E+07	1.202607E+07	-4.207454E+06							
					POINT Q	8.918537E+07	1.328559E+05	4.698598E+05	-9.459827E+04	-3.106229E+06	-3.106229E+06	7.431278E+09							
6	4	5	2		POINT P	4.326431E+07	4.288699E+04	-3.266836E+04	-3.600683E+04	1.932963E+06	1.932963E+06	2.53298E+06							
					POINT Q	-4.326431E+07	-4.288699E+04	3.266836E+04	3.600683E+04	-3.011266E+06	-3.011266E+06	1.372715E+05							
7	3	6	28		POINT P	9.544166E+07	-6.709352E+07	1.331067E+07	9.867145E+06	1.53139E+08	1.53139E+08	-6.569454E+09							
					POINT Q	-9.544166E+07	6.709352E+07	-1.331067E+07	-9.867145E+06	-5.775895E+08	-5.775895E+08	4.444933E+09							
8	4	7	30		POINT P	2.123405E+08	-1.216587E+07	2.881598E+06	1.298891E+07	6.276851E+08	6.276851E+08	-7.123222E+09							
					POINT Q	-2.123405E+08	1.216587E+07	-2.881598E+06	-1.298891E+07	-7.191026E+08	-7.191026E+08	6.70442E+09							
9	5	7	3		POINT P	4.862856E+07	-2.888498E+07	1.037496E+08	-1.570495E+07	-2.697087E+09	-2.697087E+09	-6.319781E+09							
					POINT Q	-4.862856E+07	2.888498E+07	-1.037496E+08	1.570495E+07	2.224705E+09	2.224705E+09	-5.712544E+09							
10	6	8	3		POINT P	-1.594600E+08	-1.567732E+08	-1.662653E+07	3.960516E+06	2.441239E+09	2.441239E+09	-4.456542E+09							
					POINT Q	1.594600E+08	1.567732E+08	1.662653E+07	-3.960516E+06	-1.444023E+09	-1.444023E+09	4.702638E+09							
11	7	10	4		POINT P	-1.873301E+08	-2.406391E+08	-5.706150E+07	1.169124E+07	2.560600E+09	2.560600E+09	-6.724742E+09							
					POINT Q	1.873301E+08	2.406391E+08	5.706150E+07	-1.169124E+07	-7.731100E+08	-7.731100E+08	7.3334149E+09							
12	8	9	11		POINT P	3.133247E+06	1.858049E+08	3.175125E+07	-3.463495E+07	-3.463495E+07	-3.463495E+07	4.031278E+09							
					POINT Q	-3.133247E+06	-1.858049E+08	-3.175125E+07	3.463495E+07	6.538586E+08	6.538586E+08	1.826832E+09							
13	9	10	12		POINT P	7.997026E+06	2.749702E+08	-4.717858E+08	-3.462681E+07	4.783456E+09	4.783456E+09	-1.824670E+09							
					POINT Q	-7.997026E+06	-2.749702E+08	4.717858E+08	3.462681E+07	9.888049E+06	9.888049E+06	4.618362E+09							
14	8	11	13		POINT P	-2.588178E+08	9.056839E+07	-1.025100E+08	9.971527E+06	5.447207E+09	5.447207E+09	4.668035E+09							
					POINT Q	2.588178E+08	-9.056839E+07	1.025100E+08	-9.971527E+06	-5.118799E+08	-5.118799E+08	6.235252E+09							
15	10	12	14		POINT P	6.288825E+07	2.540180E+08	-6.70072E+07	1.810623E+06	3.845303E+09	3.845303E+09	7.360758E+09							
					POINT Q	-6.288825E+07	-2.540180E+08	6.70072E+07	-1.810623E+06	6.908818E+07	6.908818E+07	-7.471765E+09							
16	11	12	13		POINT P	-2.305926E+08	2.755679E+08	9.268588E+06	3.861881E+07	-3.916082E+08	-3.916082E+08	5.602939E+09							
					POINT Q	2.305926E+08	-2.755679E+08	-9.268588E+06	-3.861881E+07	5.517508E+06	5.517508E+06	5.876119E+09							
17	11	13	8		POINT P	-3.288813E+08	3.223619E+08	-3.590061E+08	-4.672166E+07	5.338310E+09	5.338310E+09	-5.844829E+08							
					POINT Q	3.288813E+08	-3.223619E+08	3.590061E+08	4.672166E+07	6.011774E+09	6.011774E+09	1.080458E+10							

BAR FORCES AND MOMENTS
LOCAL COORDINATE SYSTEMS

BAR NUMBR	GLOBAL POINT NUMBERS P Q	POINT R	FORCE X	FORCE Y	FORCE Z	MOMENT X	MOMENT Y	MOMENT Z
18	12 14	10	-1.201164E+08	-3.559466E+08	-4.098429E+08	-6.442613E+07	5.843005E+09	-7.510180E+09
			1.201164E+08	3.559466E+08	4.098429E+08	6.442613E+07	7.153491E+09	-3.774610E+09
19	13 14	11	-2.999388E+08	-2.693324E+08	-3.809306E+05	6.970098E+07	-7.931317E+06	-5.443134E+09
			2.999388E+08	2.693324E+08	3.809306E+05	-6.970098E+07	2.379936E+07	-5.775776E+09
20	13 15	11	-1.115395E+09	-6.056762E+07	-1.950997E+06	-2.060758E+06	4.898821E+07	1.388506E+09
			1.115395E+09	6.056762E+07	1.950997E+06	2.060758E+06	-1.388506E+07	-1.076132E+06
21	14 15	12	3.491764E+06	3.041603E+05	-2.818253E+05	2.361462E+06	3.152152E+07	-1.920829E+07
			-3.491764E+06	-3.041603E+05	2.818253E+05	-2.361462E+06	-1.351304E+07	-2.235554E+05
22	13 16	18	-5.390495E+08	-6.193668E+08	-1.637794E+08	-5.632838E+06	3.932439E+08	-9.051805E+09
			5.390495E+08	6.193668E+08	1.637794E+08	5.632838E+06	-4.804836E+09	-1.060589E+07
23	14 17	20	1.209494E+08	-3.369852E+08	-1.485482E+08	-9.525949E+06	-1.333976E+09	3.694369E+09
			-1.209494E+08	3.369852E+08	1.485482E+08	9.525949E+06	6.108642E+09	-1.438971E+10
24	16 17	14	4.004135E+08	-1.238612E+08	-2.397152E+08	1.067887E+07	4.531354E+09	-2.230404E+09
			-4.004135E+08	1.238612E+08	2.397152E+08	-1.067887E+07	-5.454224E+09	-2.356520E+09
25	16 18	13	-6.616721E+08	-3.853367E+08	-1.367173E+08	1.075441E+07	-5.220281E+09	1.961675E+10
			6.616721E+08	3.853367E+08	1.367173E+08	-1.075441E+07	2.766746E+09	-1.89462E+10
26	17 20	14	-1.900115E+05	5.013956E+08	-2.334411E+08	-6.145715E+06	-6.291294E+09	1.437882E+10
			1.900115E+05	-5.013956E+08	2.334411E+08	6.145715E+06	-7.328809E+09	-1.491270E+10
27	19 19	16	8.352166E+06	-4.252396E+05	-1.301373E+08	-2.993428E+02	2.360546E+09	-8.956695E+06
			-8.352166E+06	4.252396E+05	1.301373E+08	2.993428E+02	-1.738260E+09	-4.436650E+06
28	19 20	17	4.199635E+08	-3.992058E+05	-9.562713E+08	1.400679E+02	9.703436E+09	4.469866E+06
			-4.199635E+08	3.992058E+05	9.562713E+08	-1.400679E+02	-1.228034E+07	-8.525796E+06
29	18 21	23	-6.020967E+08	-3.853362E+08	-1.370277E+08	-1.075808E+07	2.775726E+09	-1.149457E+10
			6.020967E+08	3.853362E+08	1.370277E+08	1.075808E+07	-5.229439E+09	-1.061677E+10
30	20 22	24	1.900115E+05	5.013946E+08	-2.334404E+08	6.144646E+06	7.337373E+09	-1.491271E+10
			-1.900115E+05	-5.013946E+08	2.334404E+08	-6.144646E+06	-6.299841E+09	-1.437876E+10
31	21 22	24	3.997941E+08	-1.233825E+08	-2.408862E+08	-1.067719E+07	-4.539310E+09	-2.193142E+09
			-3.997941E+08	1.233825E+08	2.408862E+08	1.067719E+07	5.461722E+09	-2.946481E+09
32	21 23	18	-5.394368E+08	-6.205329E+08	-1.632994E+08	5.696638E+06	-4.799573E+09	1.060623E+10
			5.394368E+08	6.205329E+08	1.632994E+08	-5.696638E+06	3.832745E+08	9.088431E+09
33	22 24	20	1.208259E+08	-3.369142E+08	-1.476248E+08	-9.648216E+06	-5.102872E+09	1.438829E+10
			-1.208259E+08	3.369142E+08	1.476248E+08	9.648216E+06	1.47511E+09	-3.696206E+09
34	24 24	22	-3.014755E+08	2.690715E+08	-4.463332E+05	-7.006383E+07	-6.637976E+06	5.437760E+09
			3.014755E+08	-2.690715E+08	4.463332E+05	7.006383E+07	2.521749E+07	-5.771682E+09

BAR FORCES AND MOMENTS
LOCAL COORDINATE SYSTEMS

BAR NUMBER	GLOBAL POINT NUMBERS			FORCE			FORCE			MOMENT			MOMENT		
	P	Q	R	X	Y	Z	X	Y	Z	X	Y	Z	X	Y	Z
35	23	25	21	POINT P	-1.119495E+09	-6.132402E+07	-1.976272E+06	2.133977E+06	4.960297E+07	-1.962748E+09					
			POINT Q	1.119495E+09	6.132402E+07	1.976272E+06	-2.133977E+06	-4.960297E+07	1.962748E+09						
36	24	25	22	POINT P	4.173026E+06	3.079259E+05	-2.867356E+05	2.345557E+06	3.195032E+07	1.936631E+07					
			POINT Q	-4.173026E+06	-3.079259E+05	2.867356E+05	-2.345557E+06	-3.195032E+07	-1.936631E+07						
37	23	26	20	POINT P	3.298882E+08	-3.246212E+08	3.593704E+08	4.690425E+07	-6.028253E+09	-1.086276E+10					
			POINT Q	3.298882E+08	3.246212E+08	-3.593704E+08	-4.690425E+07	6.028253E+09	1.086276E+10						
38	24	27	20	POINT P	-1.199927E+08	3.560894E+08	4.101261E+08	-6.470065E+07	-7.171846E+09	3.776773E+09					
			POINT Q	1.199927E+08	-3.560894E+08	-4.101261E+08	6.470065E+07	7.171846E+09	-3.776773E+09						
39	26	27	24	POINT P	-2.293966E+08	2.759769E+08	-1.055208E+07	-3.878545E+07	4.168584E+08	5.612745E+09					
			POINT Q	2.293966E+08	-2.759769E+08	1.055208E+07	3.878545E+07	-4.168584E+08	-5.612745E+09						
40	26	28	23	POINT P	-2.599978E+08	-9.056280E+07	-1.036847E+08	1.006008E+07	-5.516229E+09	-4.681298E+09					
			POINT Q	2.599978E+08	9.056280E+07	1.036847E+08	-1.006008E+07	5.516229E+09	4.681298E+09						
41	27	30	24	POINT P	6.362890E+07	-2.537659E+08	6.831450E+07	-1.929124E+06	-1.020316E+08	-7.473612E+09					
			POINT Q	-6.362890E+07	2.537659E+08	-6.831450E+07	1.929124E+06	1.020316E+08	7.473612E+09						
42	28	29	1	POINT P	3.096410E+06	-1.873529E+08	3.232333E+07	-3.583492E+07	-3.556837E+08	-4.063310E+09					
			POINT Q	-3.096410E+06	1.873529E+08	-3.232333E+07	3.583492E+07	3.556837E+08	4.063310E+09						
43	29	30	2	POINT P	8.825360E+06	-2.763893E+08	-4.702663E+08	3.542690E+07	4.767884E+09	1.844662E+09					
			POINT Q	-8.825360E+06	2.763893E+08	4.702663E+08	-3.542690E+07	-4.767884E+09	-1.844662E+09						
44	1	28	3	POINT P	-1.590604E+08	-1.572479E+08	-1.695516E+07	4.231039E+06	-2.438808E+09	-4.716782E+09					
			POINT Q	1.590604E+08	1.572479E+08	1.695516E+07	-4.231039E+06	2.438808E+09	4.716782E+09						
45	2	30	4	POINT P	-1.079883E+08	-2.394464E+08	5.733671E+07	-1.193982E+07	-2.589172E+09	-6.672427E+09					
			POINT Q	1.079883E+08	2.394464E+08	-5.733671E+07	1.193982E+07	2.589172E+09	6.672427E+09						
46	28	31	8	POINT P	-1.848152E+08	1.228681E+08	-8.648553E+07	4.863881E+06	3.498194E+08	0.					
			POINT Q	1.848152E+08	-1.228681E+08	8.648553E+07	-4.863881E+06	-3.498194E+08	0.						
47	29	32	9	POINT P	-4.738507E+08	1.140176E+08	-3.751545E+06	3.949656E+06	-4.105504E+09	0.					
			POINT Q	4.738507E+08	-1.140176E+08	3.751545E+06	-3.949656E+06	4.105504E+09	0.						
48	8	31	18	POINT P	-1.472860E+08	-1.229060E+08	-8.610530E+07	-4.789116E+06	3.401309E+08	0.					
			POINT Q	1.472860E+08	1.229060E+08	8.610530E+07	4.789116E+06	-3.401309E+08	0.						
49	9	32	19	POINT P	-4.747808E+08	-1.141033E+08	-3.121859E+06	3.880420E+06	-4.129986E+09	0.					
			POINT Q	4.747808E+08	1.141033E+08	3.121859E+06	-3.880420E+06	4.129986E+09	0.						
50	18	31	8	POINT P	7.529094E+08	-7.995244E+03	1.802733E+08	-2.493369E+04	-2.382027E+09	0.					
			POINT Q	-7.529094E+08	7.995244E+03	-1.802733E+08	2.493369E+04	2.382027E+09	0.						
51	19	32	9	POINT P	7.533622E+08	-4.417866E+04	3.488801E+08	-2.318847E+04	-1.144471E+10	0.					
			POINT Q	-7.533622E+08	4.417866E+04	-3.488801E+08	2.318847E+04	1.144471E+10	0.						

BAR FORCES AND MOMENTS
LOCAL COORDINATE SYSTEMS

BAR NUMER	MODAL POINT NUMBERS			FORCE			MOMENT		
	P	R	S	X	Y	Z	X	Y	Z
52	31	33	3	POINT P -7.695605E+06	-2.562079E+08	1.041596E+06	-3.613494E+06	-1.362884E+07	-4.584080E+09
			POINT Q	7.695605E+06	2.562079E+08	-1.041596E+06	3.613494E+06	-5.254138E+05	1.102473E+03
53	32	33	4	POINT P 2.786800E+07	-3.852027E+08	1.105555E+06	-2.678310E+06	-1.756909E+07	8.752367E+09
			POINT Q	-2.786800E+07	-3.852027E+08	-1.105555E+06	2.678310E+06	-2.228071E+06	-1.854543E+09

CPU TIME USAGE TABLE (SEC)

	TIME IN	TIME OUT	TOTAL
INPUT AND INITIALIZATION OVERLAY	.347	1.363	1.710
STRUCTURAL STIFFNESS MAT-RX GENERATION OVERLAY	1.363	2.465	3.828
DISPLACEMENT, ROTATION, FORCE, AND MOMENT SOLUTION OVERLAY	2.465	42.421	44.886

APPENDIX B

EXAMPLE OF INPUT AND OUTPUT

MODAL ANALYSIS ROUTINE

STRUCTURAL ANALYSIS PROGRAM

STRUCTURAL ANALYSIS PROGRAM --- LEGGED LANDER
 MASTER AGREEMENT, CONTRACT NAS1-8137, TASK ORDER NUMBER FIVE
 MCDONNELL DOUGLAS AERONAUTICS COMPANY, EAST

STRUCTURAL ANALYSIS DATA - CARD CODE

BLANK - 0	COMMENTS
1	NODAL POINT DEFINITIONS
2	REFERENCE POINTS
3	NODAL POINT RESTRAINT DEFINITIONS
4	FORCE VECTORS
5	MOMENT VECTORS
6	BAR DEFINITIONS
7	SHEAR PANEL DEFINITIONS
8	FORMAIED-DATA TERMINATOR
1 13.5890	31.7246 85.7758
1 28.0670	31.7246 85.7758
1 13.5890	0. 85.7758
1 28.0670	0. 85.7758
1 29.4894	0. 113.538
1 13.5890	-31.7245 85.7758
1 28.0670	-31.7245 85.7758
1 13.5890	-60.9346 35.1790
1 17.9070	-60.9346 35.1790
1 28.0670	-60.9346 35.1790
1 13.5890	-90.1446 -15.4174
1 28.0670	-90.1446 -15.4174
1 13.5890	-74.2950 -42.8752
1 28.0670	-74.2950 -42.8752
1 29.4894	-98.3234 -56.7690
1 13.5890	-58.42 -70.358
1 28.0670	-58.42 -70.358
1 13.5890	0. -70.358
1 17.9070	0. -70.358
1 28.0670	0. -70.358
1 13.5890	58.42 -70.358
1 28.0670	58.42 -70.358
1 13.5890	74.295 -42.8752
1 28.0670	74.295 -42.8752
1 29.4894	98.3234 -56.769
1 13.5890	90.1446 -15.4178
1 28.0670	90.1446 -15.4178
1 13.5890	50.9346 35.179
1 17.9070	50.9346 35.179
1 28.0670	50.9346 35.179
1 13.5890	0. 0.
1 17.9070	0. 0.
1 28.0670	0. 0.
1 1 2 3	11.7742 106.70 168.10
1 2 3 4	16.1290 111.43 224.90
1 3 4 30	24.8322 435.21 700.43
1 4 3 4 1	14.1129 110.95 4.87
1 5 3 5 1	7.6613 55.53 2.22
1 6 4 5 2	4.8387 6.98 3.68
1 7 3 6 28	16.1290 111.43 274.90
1 8 4 7 30	24.8322 435.21 700.43
1 6 7 3	11.7742 106.70 168.10
1 7 3	9.2774 29.58 4010.2
1 8 3	9.2774 29.58 4010.2
1 9 11	20.729 255.28 109.46
1 10 12	16.3742 253.57 27.46
1 11 13	9.2774 29.58 4010.2
1 12 13	1.5775 6.826E11 2.620E11
1 13 13	2.1644 6.826E11 2.620E11
1 14 13	3.3299 6.826E11 2.620E11
1 15 13	6.4557 6.826E11 2.620E11
1 16 13	1.0261 6.826E11 2.620E11
1 17 13	6.2431 6.826E11 2.620E11
1 18 13	2.1644 6.826E11 2.620E11
1 19 13	3.3299 6.826E11 2.620E11
1 20 13	1.5775 6.826E11 2.620E11
1 21 13	1.2445 6.826E11 2.620E11
1 22 13	1.2445 6.826E11 2.620E11
1 23 13	2.7804 6.826E11 2.620E11
1 24 13	2.1977 6.826E11 2.620E11
1 25 13	1.2445 6.826E11 2.620E11

6	15	10	12	14	9.2774	29.58	4010.2	1.2445	6.826E11	2.620E11
6	16	11	12	13	11.7742	106.70	168.10	1.5775	6.826E11	2.620E11
6	17	11	13	3	16.1290	111.43	274.90	2.1644	6.826E11	2.620E11
6	18	12	14	10	24.8322	435.21	700.43	3.3299	6.826E11	2.620E11
6	19	13	14	11	14.1129	110.95	4.87	6.4557	6.826E11	2.620E11
6	20	13	15	11	7.6613	55.53	2.22	1.0281	6.826E11	2.620E11
6	21	14	15	12	4.9387	6.98	3.68	.6243	6.826E11	2.620E11
6	22	13	16	18	16.1290	111.43	274.90	2.1644	6.826E11	2.620E11
6	23	14	17	20	24.8322	435.21	700.43	3.3299	6.826E11	2.620E11
6	24	16	17	14	11.7742	106.70	168.10	1.5775	6.826E11	2.620E11
6	25	16	18	13	9.2774	29.58	4010.2	1.2445	6.826E11	2.620E11
6	26	17	20	14	3.2774	29.58	4010.2	1.2445	6.826E11	2.620E11
6	27	18	19	16	20.7290	259.28	109.46	2.7804	6.826E11	2.620E11
6	28	19	20	17	16.3742	253.57	27.46	2.1977	6.826E11	2.620E11
6	29	18	21	23	9.2774	29.58	4010.2	1.2445	6.826E11	2.620E11
6	30	20	22	24	9.2774	29.58	4010.2	1.2445	6.826E11	2.620E11
6	31	21	22	24	11.7742	106.70	168.10	1.5775	6.826E11	2.620E11
6	32	21	23	18	16.1290	111.43	274.90	2.1644	6.826E11	2.620E11
6	33	22	24	20	24.8322	435.21	700.43	3.3299	6.826E11	2.620E11
6	34	23	24	22	14.1129	110.95	4.87	6.4557	6.826E11	2.620E11
6	35	23	25	21	7.6613	55.53	2.22	1.0281	6.826E11	2.620E11
6	36	24	25	22	4.9387	6.98	3.68	.6243	6.826E11	2.620E11
6	37	23	25	28	16.1290	111.43	274.90	2.1644	6.826E11	2.620E11
6	38	24	27	30	24.8322	435.21	700.43	3.3299	6.826E11	2.620E11
6	39	26	27	24	11.7742	106.70	168.07	1.5775	6.826E11	2.620E11
6	40	26	28	23	9.2774	29.58	4010.2	1.2445	6.826E11	2.620E11
6	41	27	30	24	9.2774	29.58	4010.2	1.2445	6.826E11	2.620E11
6	42	28	29	1	20.7290	255.28	109.46	2.7804	6.826E11	2.620E11
6	43	29	30	2	16.3742	253.57	27.46	2.1977	6.826E11	2.620E11
6	44	1	28	3	9.2774	29.58	4010.2	1.2445	6.826E11	2.620E11
6	45	2	30	4	9.2774	29.58	4010.2	1.2445	6.826E11	2.620E11
6	46	26	31	8	1.8.0645	47.28	2004.2	1.0822	6.826E11	2.620E11
6	47	29	32	9	1.8.0645	47.28	2004.2	1.0822	6.826E11	2.620E11
6	48	8	31	18	1.8.0645	47.28	2004.2	1.0822	6.826E11	2.620E11
6	49	9	32	19	1.8.0645	47.28	2004.2	1.0822	6.826E11	2.620E11
6	50	18	31	8	1.8.0645	47.28	2004.2	1.0822	6.826E11	2.620E11
6	51	19	32	9	1.8.0645	47.28	2004.2	1.0822	6.826E11	2.620E11
6	52	31	33	3	58.7741	2776.3	2776.3	5551.7	6.826E11	2.620E11
6	53	32	33	4	58.7741	2776.3	2776.3	5551.7	6.826E11	2.620E11

STRUCTURAL ANALYSIS CONTROL DATA

GENERAL DATA

INPKT = 0 ; 1 IMPLIES READ ALL MATRICES FROM TAPE
 INDSFG = 1 ; 0 IMPLIES WRITE GLOBAL BAR MATRICES
 INDSFL = 1 ; 0 IMPLIES WRITE LOCAL BAR MATRICES
 INDMKT = 0 ; 1 IMPLIES SAVE ALL MATRICES ON TAPE
 INDMTS = 0 ; 1 IMPLIES PRINT TOTAL STIFFNESS MATRIX
 ISEDIM = 12004 ; MAX. STORAGE FOR STIFFNESS MATRIX

MODAL ANALYSIS DATA

INONMA = 1 ; RUN NORMAL MODE ANALYSIS
 INONNM = 1 ; 1 IMPLIES WRITE MODE DATA ON TAPE
 IPEDTO = 99 ; ORDER OF REDUCE SYSTEM
 NFIGVL = 20 ; REQUIRED NON-RIGID BODY MODES

AMASS IS THE DIAGONAL MASS VECTOR OF THE REDUCE SYSTEM

3.799E+03	3.799E+03	3.799E+03	6.099E+03	6.099E+03	3.549E+03	3.549E+03	3.549E+03	2.490E+03
2.490E+03	2.490E+03	3.483E+03	3.483E+03	3.483E+03	3.789E+03	3.789E+03	3.789E+03	6.099E+03
6.099E+03	3.533E+03	3.533E+03	3.533E+03	1.879E+03	1.879E+03	1.736E+03	1.736E+03	1.736E+03
3.789E+03	3.789E+03	3.789E+03	6.099E+03	6.099E+03	3.549E+03	3.549E+03	3.549E+03	2.127E+03
2.127E+03	2.127E+03	3.483E+03	3.483E+03	3.483E+03	3.789E+03	3.789E+03	6.099E+03	6.099E+03
6.099E+03	3.812E+03	3.812E+03	3.812E+03	2.346E+03	2.346E+03	1.736E+03	1.736E+03	1.736E+03
3.789E+03	3.789E+03	3.789E+03	6.099E+03	6.099E+03	3.549E+03	3.549E+03	3.549E+03	2.127E+03
2.127E+03	2.127E+03	3.483E+03	3.483E+03	3.483E+03	3.789E+03	3.789E+03	6.099E+03	6.099E+03
6.099E+03	3.533E+03	3.533E+03	3.533E+03	1.879E+03	1.879E+03	1.736E+03	1.736E+03	1.736E+03
3.487E+04	3.487E+04	3.487E+04	3.270E+04	3.270E+04	3.578E+03	3.578E+03	3.578E+03	3.578E+03

IPKRC CONTAINS THE ROWS TO KEEP IN THE REDUCED SYSTEM

1	2	3	7	8	9	13	14	15
20	21	25	26	27	31	26	32	37
39	43	44	45	49	50	51	55	57
61	62	63	67	68	69	73	74	75
80	81	85	86	87	91	92	93	97
99	103	104	105	109	110	111	115	116
121	122	123	127	128	129	133	134	135
140	141	145	146	147	151	152	153	157
159	163	164	165	169	170	171	175	176
181	182	183	187	188	189	193	194	195

GENERALIZED INERTIA PROPERTIES

VARIABLES	1	2	3	4	5
WNY	9.40331315E+04	1.12127882E+05	2.9685087E+04	3.42179713E+04	5.06333976E+04
WNY	5.43594868E+04	2.66459996E+04	5.37421E+04	5.16372988E+04	7.68004646E+04
WNY	1.97048967E+04	8.55226510E+04	5.41735060E+04	5.14136396E+04	7.68867378E+04
PX	2.43423465E+01	-3.53679722E-01	4.67697E21E+03	-2.57496930E+02	-1.07756323E+02
PY	-2.42335120E+06	-1.66506291E+02	-1.10336184E+05	1.90101818E+03	4.44512439E+01
PZ	2.48351470E+06	1.70546160E+02	1.00304156E+05	-1.55043815E+03	2.11936905E+02
GM DIAGONAL	9.40497575E+04	1.12148266E+05	6.88004331E+04	7.86644454E+04	1.07190300E+05

NATURAL FREQUENCIES (RAD/SEC)

1	2	3	4	5
2.17351330E+02	2.17377127E+02	3.0605553E+02	3.06529126E+02	3.94729713E+02

CORRESPONDING MODE SHAPES FOR COMPLETE SYSTEM

MODAL POINT NUMBER	ELEMENT REFERENCE	1	2	3	4	5
1	X-TRANS.	-7.6552899E-04	-2.44301231E-03	-3.9857563E-01	-9.87226277E-01	-3.79674797E-01
1	Y-TRANS.	-3.09653265E-04	-6.04528241E-03	9.66601179E-03	6.06508516E-01	4.87913279E-01
1	Z-TRANS.	8.73360464E-01	-5.06504763E-01	2.97151277E-01	5.04744526E-01	5.92113821E-01
2	X-TRANS.	-7.58548871E-04	-2.42282525E-03	-3.99066798E-01	-9.87226277E-01	-8.79674797E-01
2	Y-TRANS.	-4.30481914E-04	-1.26662378E-02	-9.8095249E-03	-5.37463958E-01	-6.59620596E-01
2	Z-TRANS.	8.72368550E-01	-5.89846423E-01	-2.95166740E-01	-5.92420959E-01	-7.69105691E-01
3	X-TRANS.	-2.59211498E-04	3.97425501E-07	-3.86296660E-01	6.399902676E-03	1.94579946E-04
3	Y-TRANS.	4.31750705E-07	-7.93330387E-03	8.56827118E-03	6.08272506E-01	4.88943089E-01
3	Z-TRANS.	9.67557094E-01	6.63031535E-05	3.15078539E-01	-5.10705596E-03	-6.556310569E-05
4	X-TRANS.	-7.77723368E-04	3.62913769E-07	-3.87278979E-01	6.44177494E-03	1.95392954E-04
4	Y-TRANS.	5.36824826E-07	-1.26567091E-02	-9.9312775E-03	-5.38686131E-01	-6.59620596E-01
4	Z-TRANS.	8.97323894E-01	6.16871752E-05	-2.92485269E-01	4.75394136E-03	7.00313008E-05
5	X-TRANS.	-5.36389826E-02	-3.12289330E-06	-8.11640472E-01	1.33029197E-02	2.36205962E-04
5	Y-TRANS.	-2.44923047E-05	5.9956363E-01	9.79616896E-03	6.96472019E-01	4.8323035E-01
5	Z-TRANS.	1.00000000E+00	6.88475132E-05	5.62131631E-01	-9.11800487E-03	-1.25874235E-04
6	X-TRANS.	-7.646117519E-04	2.44374196E-03	-3.70579168E-01	1.00000000E+00	8.79674797E-01
6	Y-TRANS.	3.1826176E-04	-8.882447E-03	7.4214154E-03	5.06569343E-01	4.87913279E-01
6	Z-TRANS.	8.73388845E-01	5.88824439E-01	2.82662083E-01	-5.14411922E-01	-5.82113821E-01
7	X-TRANS.	-7.58582688E-04	2.42355799E-03	-3.74070727E-01	1.00000000E+00	8.80216802E-01
7	Y-TRANS.	4.3176741E-04	-1.26661782E-02	-6.35216110E-03	-6.37469586E-01	-6.59620596E-01
7	Z-TRANS.	8.72316169E-01	5.89166364E-01	-2.79960707E-01	5.41788321E-01	7.69647696E-01
8	X-TRANS.	-1.59738033E-03	1.48958166E-03	3.22691552E-01	5.74513382E-01	-3.8272829E-03
8	Y-TRANS.	6.42685210E-01	-2.82964038E-01	2.81428361E-01	6.77007299E-02	-1.31327913E-01
8	Z-TRANS.	6.99994558E-01	7.46027047E-01	1.64783890E-01	-2.44598540E-01	-2.28988899E-01
9	X-TRANS.	-1.56067247E-03	1.33559793E-03	3.22937132E-01	5.74939173E-01	-3.82981030E-03
9	Y-TRANS.	6.42781317E-01	-2.82951430E-01	-1.20142967E-01	-6.30170316E-02	9.75609756E-02
9	Z-TRANS.	6.99560899E-01	7.45567299E-01	-7.31090373E-02	1.32526299E-01	1.63794038E-01
10	X-TRANS.	-1.57412693E-03	1.37163774E-03	3.22691552E-01	5.74391727E-01	-3.82721002E-03
10	Y-TRANS.	6.39983011E-01	-2.76640912E-01	-2.35584479E-01	-1.25729927E-01	1.73004130E-01
10	Z-TRANS.	5.81605094E-01	7.42150279E-01	-1.83666047E-01	2.56751825E-01	2.93333333E-01
11	X-TRANS.	-2.95658276E-03	1.51243315E-03	9.99754420E-01	1.20437956E-01	-8.87262873E-01
11	Y-TRANS.	7.68073416E-01	4.94111593E-01	1.29371316E-01	1.28832117E-01	-7.53929399E-01
11	Z-TRANS.	4.32949065E-01	3.80239355E-01	-1.46782908E-01	-2.60888078E-01	1.35013550E-01
12	X-TRANS.	-2.94442421E-03	1.49663207E-03	1.00000000E+00	1.20012165E-01	-8.87804879E-01

CORRESPONDING NODE SHAPES FOR COMPLETE SYSTEM

NODAL POINT NUMBER	ELEMENT REFERENCE					
	1	2	3	4	5	
12	Y-TRANS.	7.64954158E-01	4.90792255E-01	-7.57367297E-01	-1.42274939E-01	1.00000000E+00
12	Z-TRANS.	4.29695377E-01	3.00832978E-01	1.63752456E-01	2.76094691E-01	-1.86617896E-01
13	X-TRANS.	1.10464596E-04	-2.44433329E-05	1.93393910E-01	-3.61961314E-01	-3.51327913E-05
13	Y-TRANS.	4.21694583E-01	8.28495119E-01	3.79665012E-01	-1.04987835E-01	-2.45203252E-01
13	Z-TRANS.	2.37541799E-01	4.92840928E-01	-3.49705705E-01	-1.98114359E-01	4.29311969E-01
14	X-TRANS.	3.70019353E-04	4.93961797E-04	1.93934135E-01	-1.62895377E-01	-2.62113821E-05
14	Y-TRANS.	3.93843760E-01	7.65952249E-01	-8.3940864E-01	9.39807796E-02	3.32945294E-01
14	Z-TRANS.	2.15227415E-01	4.59153826E-01	3.79666012E-01	4.07846715E-01	-5.74525745E-01
15	X-TRANS.	2.69010444E-02	5.38564987E-02	4.00540275E-01	-7.58515915E-01	1.58590796E-03
15	Y-TRANS.	2.26490270E-01	9.95800024E-01	5.18172898E-01	-2.94124089E-01	-2.44119421E-01
15	Z-TRANS.	6.07667741E-01	2.68734323E-01	-3.44969566E-01	-5.48965937E-01	4.24769648E-01
16	X-TRANS.	3.60915742E-03	-5.43844143E-04	-6.15667976E-01	-9.41240876E-01	9.48346893E-01
16	Y-TRANS.	1.45326090E-03	1.00000000E+00	6.42834500E-03	-2.90635038E-01	2.64878049E-01
16	Z-TRANS.	-5.48291723E-03	5.91686768E-01	-5.63113949E-01	-5.10765423E-01	7.23035230E-01
17	X-TRANS.	3.66975199E-03	-5.38010650E-04	-6.15422377E-01	-9.41949144E-01	9.48346883E-01
17	Y-TRANS.	9.66003308E-04	9.98522579E-01	-5.33399821E-03	3.02990535E-01	-3.35392954E-01
17	Z-TRANS.	-1.19730161E-02	5.92989664E-01	5.97249509E-01	5.34549878E-01	-9.59991599E-01
18	X-TRANS.	3.14894165E-03	1.22584663E-07	-6.19597250E-01	9.92700730E-03	1.81182412E-05
18	Y-TRANS.	-4.45210559E-05	9.98224417E-01	-8.20432220E-03	-2.99939179E-01	2.68726287E-01
18	Z-TRANS.	-6.13274776E-01	-4.21099122E-05	-1.79543222E-01	2.86678832E-03	-3.95663957E-06
19	X-TRANS.	3.07529500E-03	1.17732579E-07	-6.20089409E-01	3.93309029E-03	1.91734447E-05
19	Y-TRANS.	-4.44745850E-05	9.97295510E-01	2.18544257E-03	1.25778599E-01	-1.88583696E-01
19	Z-TRANS.	-5.13914779E-01	-4.20564779E-05	1.33655206E-01	-2.09975669E-03	4.78102998E-06
20	X-TRANS.	3.11015996E-03	1.20220086E-07	-6.19597250E-01	9.92700730E-03	1.80487805E-05
20	Y-TRANS.	-4.44818915E-05	9.97606258E-01	4.25569391E-03	3.03406326E-01	-3.39241192E-01
20	Z-TRANS.	-6.07007874E-01	-4.15882996E-05	2.58104126E-01	-4.11922141E-03	5.31544715E-06
21	X-TRANS.	3.60817926E-03	5.444140590E-04	-5.91946759E-01	3.60705996E-01	-3.88346893E-01
21	Y-TRANS.	-1.54247899E-03	9.99999779E-01	-1.48747544E-02	-2.99330900E-01	2.64878049E-01
21	Z-TRANS.	-5.43011646E-03	-5.91607730E-01	-5.44879175E-01	5.26598008E-01	-7.23035230E-01
22	X-TRANS.	3.66976496E-03	5.38316534E-04	-5.91501179E-01	9.61313863E-01	-8.88346883E-01
22	Y-TRANS.	-1.03585610E-03	9.98224417E-01	1.38285859E-02	3.06678599E-01	-1.88583696E-01
22	Z-TRANS.	-1.19202667E-02	-5.92990972E-01	5.82023576E-01	-5.55527981E-01	9.59991599E-01
23	X-TRANS.	1.10512526E-04	-2.43401510E-05	2.03487230E-01	-3.554013629E-01	-3.699051491E-05
23	Y-TRANS.	-4.21768507E-01	8.28437206E-01	-3.82367397E-01	9.27007299E-02	-2.45094851E-01
23	Z-TRANS.	2.37595808E-01	-4.92808538E-01	-3.38409644E-01	4.09124088E-01	-4.29364499E-01
24	X-TRANS.	3.70146751E-04	-4.93378346E-04	2.04075621E-01	3.56447699E-01	-4.64281893E-05
24	Y-TRANS.	-3.93912050E-01	7.65898444E-01	3.86051691E-01	7.14720195E-01	3.32791328E-01
24	Z-TRANS.	2.152268216E-01	-4.59123979E-01	3.68123772E-01	-4.19829684E-01	5.74525745E-01
25	X-TRANS.	2.69063978E-02	-5.38327712E-02	4.21660118E-01	7.45133820E-01	-1.75609756E-03

CORRESPONDING NODE SHAPES FOR COMPLETE SYSTEM

NODAL POINT NUMBER	ELEMENT REFERENCE	5				
		1	2	3	4	5
25	Y-TRANS.	-2.25573147E-01	9.95848202E-01	-5.25785258E-01	-2.67274939E-01	-2.40975610E-01
25	Z-TRANS.	5.37692423E-01	-2.68652123E-01	-3.33497153E-01	5.59954015E-01	-4.24769648E-01
26	X-TRANS.	-2.95637365E-03	-1.51277693E-03	9.96070727E-01	-1.522554745E-01	9.87262873E-01
26	Y-TRANS.	-7.50117433E-01	4.94007192E-01	-7.25687623E-01	1.52311436E-01	-7.53929539E-01
26	Z-TRANS.	4.32975876E-01	-3.00180144E-01	-1.39445212E-01	2.65450122E-01	-1.35013550E-01
27	X-TRANS.	-2.94421518E-03	-1.49690350E-03	9.96316506E-01	-1.52128954E-01	9.87262873E-01
27	Y-TRANS.	-7.54397375E-01	4.90687338E-01	7.52946959E-01	-1.66405839E-01	1.00000000E+00
27	Z-TRANS.	4.2672261E-01	-3.00773716E-01	1.55943026E-01	-2.81204380E-01	1.88672087E-01
28	X-TRANS.	-1.59722890E-03	-1.40936794E-03	3.06483301E-01	-5.84489051E-01	3.93550136E-03
28	Y-TRANS.	-5.42559981E-01	-2.83092187E-01	-1.9949725E-01	7.42092457E-02	-1.31327913E-01
28	Z-TRANS.	5.00061108E-01	-7.459958561E-01	1.70776031E-01	2.09184915E-01	2.26790488E-01
29	X-TRANS.	-1.56059758E-03	-1.33537968E-03	3.06729880E-01	-5.844854015E-01	3.93712737E-03
29	Y-TRANS.	-6.42755056E-01	-2.83039644E-01	1.18295678E-01	-6.69099757E-02	9.75067751E-02
29	Z-TRANS.	4.99627302E-01	-7.45498597E-01	-7.68172889E-02	-1.30948662E-01	-1.63739837E-01
30	X-TRANS.	-1.57804869E-03	-1.37142237E-03	3.06483301E-01	-5.84306569E-01	3.93387534E-03
30	Y-TRANS.	-6.39959309E-01	-2.76728754E-01	2.31974460E-01	-1.33272506E-01	1.7295930E-01
30	Z-TRANS.	5.01671159E-01	-7.42081213E-01	-1.50834971E-01	-2.51946472E-01	-2.93224932E-01
31	X-TRANS.	-2.03274563E-05	1.16369554E-07	3.48722986E-03	-3.941610584E-06	4.75176153E-05
31	Y-TRANS.	3.19899868E-05	-7.17412588E-01	2.52946059E-03	1.80352798E-01	9.40379404E-04
31	Z-TRANS.	-6.16432244E-01	-4.23244294E-05	-1.67387133E-01	2.67153285E-03	-4.68997290E-06
32	X-TRANS.	-2.02422691E-05	1.16366734E-07	3.48722086E-03	-3.93917275E-06	4.75067751E-05
32	Y-TRANS.	3.20053588E-05	-7.18027219E-01	-1.77725933E-03	-1.26703163E-01	3.00379404E-03
32	Z-TRANS.	-6.17769949E-01	-4.23227614E-05	1.18492141E-01	-1.88807786E-03	5.52845528E-06
33	X-TRANS.	-2.02913627E-05	1.16372161E-07	3.48722046E-03	-3.94039299E-06	4.75176153E-05
33	Y-TRANS.	3.19955994E-05	-7.17655504E-01	6.59941061E-04	4.77554745E-02	1.8303230E-03
33	Z-TRANS.	-6.16989651E-01	-4.23223351E-05	-4.39096267E-02	7.01946472E-04	-2.99574797E-07

GENERALIZED INERTIA PROPERTIES

VARIABLES	6	7	8	9	10
MX	4.95076507E+04	6.10583024E+04	5.03085030E+04	1.07689532E+04	5.77606280E+03
MY	2.61634448E+04	2.30648016E+04	3.69418679E+04	8.14502530E+03	1.80811712E+03
MZ	2.63276932E+04	3.91984612E+04	2.17051152E+04	3.36052430E+03	4.37789279E+03
PX	1.91219536E+01	-1.60387525E+01	1.83574012E+02	-2.78692195E+02	1.22824239E+01
PY	-2.56588463E+01	4.46301049E+01	-2.33262960E+06	3.80844061E+05	-9.50505703E+02
PZ	1.83602271E+01	-2.07531475E+01	2.32055734E+06	-3.00791830E+05	8.95655019E+02
GM DIAGONAL	5.18993903E+04	6.20608226E+04	5.84775534E+04	1.11472515E+04	5.90103636E+03

NATURAL FREQUENCIES (RADIANS/SEC)

5.42350223E+02	7	3	9	10
6.33268051E+02	6.34553229E+02	3.57393833E+02	9.57550540E+02	

CORRESPONDING MODE SHAPES FOR COMPLETE SYSTEM

MODAL POINT NUMBER	ELEMENT REFERENCE	6	7	8	9	10
1	X-TRANS.	1.57879377E-01	-7.51736915E-02	3.30368268E-03	8.98575551E-03	7.02331689E-02
1	Y-TRANS.	-4.35038911E-01	9.96294591E-01	3.3891310E-03	-1.48051912E-03	-1.22433954E-01
1	Z-TRANS.	-3.68871595E-01	6.68828161E-01	2.89689339E-01	-3.88862737E-02	-7.38441498E-02
2	X-TRANS.	1.58268442E-01	-7.47105141E-02	3.19358913E-03	9.15589956E-03	7.24122997E-02
2	Y-TRANS.	-2.95428016E-01	9.09680438E-01	1.92470179E-03	-5.2525315E-04	-4.64703335E-02
2	Z-TRANS.	-6.28599222E-01	6.10930987E-01	1.88048649E-01	-8.18051403E-02	5.5738449E-03
3	X-TRANS.	-3.05155642E-05	1.37795276E-05	3.96034898E-03	1.02342698E-02	-3.13469034E-05
3	Y-TRANS.	-4.95330739E-01	1.0000099E+00	1.76920282E-05	-5.44795668E-04	-1.18406236E-01
3	Z-TRANS.	1.88949416E-05	-9.129222549E-06	1.82934007E-01	-8.71387603E-02	2.21697705E-04
4	X-TRANS.	-3.87782101E-05	1.39740621E-05	3.05265527E-03	1.12728900E-02	-3.43915115E-05
4	Y-TRANS.	-2.95525292E-01	9.18606762E-01	1.45882489E-05	-2.13721030E-04	-4.65569511E-02
4	Z-TRANS.	-1.10603113E-05	2.35942856E-06	1.88772600E-01	-8.50590502E-02	2.01882720E-04
5	X-TRANS.	-4.86673152E-05	2.44435337E-05	-1.71021818E-02	2.81262047E-02	-9.17280208E-05
5	Y-TRANS.	3.88618677E-01	6.37332038E-01	1.1600049E-05	4.60302705E-03	1.0000000E+00
5	Z-TRANS.	2.17396109E-05	-1.54284391E-05	1.98224658E-01	-1.818172008E-01	2.67345171E-04
6	X-TRANS.	-1.57879377E-01	7.51736915E-02	3.30847024E-03	8.33878340E-03	-7.03334777E-02
6	Y-TRANS.	-4.95038911E-01	9.96294591E-01	-3.15315530E-03	3.53888208E-04	-1.22433954E-01
6	Z-TRANS.	8.68871595E-01	-6.68828161E-01	2.09665747E-01	-3.818748906E-02	7.40147250E-02
7	X-TRANS.	-1.58365759E-01	7.47568319E-02	3.1983282E-03	8.48953742E-03	-7.24555085E-02
7	Y-TRANS.	-2.95428016E-01	9.09680438E-01	-1.99959532E-03	9.88672423E-05	-4.64703335E-02
7	Z-TRANS.	5.28599222E-01	-6.10930987E-01	1.88828893E-01	-8.18051403E-02	-5.18406236E-03
8	X-TRANS.	-1.49027237E-03	5.62760537E-02	2.95292720E-02	4.52475437E-02	-5.83402512E-02
8	Y-TRANS.	2.54065603E-01	-2.83001390E-01	-4.35471824E-02	-6.93341614E-03	-1.31485492E-02
8	Z-TRANS.	4.39883268E-01	4.37702640E-02	2.21873272E-01	-2.55866177E-02	-5.06712863E-03
9	X-TRANS.	-1.49027237E-03	5.62760537E-02	2.91989828E-02	4.53236128E-02	-5.84663699E-02
9	Y-TRANS.	2.13424125E-01	-2.98749421E-01	-5.99773294E-02	-4.24327294E-02	4.12906020E-03
9	Z-TRANS.	3.68190661E-01	6.35479319E-02	2.16770827E-01	-4.394905989E-02	-3.13555652E-02
10	X-TRANS.	-1.48832695E-03	5.59591941E-02	2.93019615E-02	4.52357110E-02	-5.83363424E-02
10	Y-TRANS.	2.00000000E-01	-2.49143122E-01	-3.48471826E-02	-5.38991126E-02	7.65565613E-03
10	Z-TRANS.	3.44455253E-01	3.86521538E-02	1.97230224E-01	-4.92159568E-02	-3.76483326E-02
11	X-TRANS.	1.55058366E-01	3.70819824E-02	5.57177430E-02	7.78242699E-02	-4.11887051E-02
11	Y-TRANS.	1.00000000E+00	3.76609541E-01	1.00000000E+00	1.30315397E-01	-9.51017577E-02
11	Z-TRANS.	6.25689934E-03	-3.60676239E-01	-3.97289132E-01	-9.06719100E-02	2.30142919E-02
12	X-TRANS.	1.55544474E-01	3.67531254E-02	5.83835539E-02	8.02499240E-02	-4.22367042E-02

CORRESPONDING MODE SHAPES FOR COMPLETE SYSTEM

NODAL POINT NUMBER	ELEMENT REFERENCE	10													
		6	7	8	9	10	11	12	13	14	15				
12	Y-TRANS.	5.944552535E-01	3.559591344E-01	9.205641066E-01	-1.192036099E-02	-5.491554799E-02									
12	Z-TRANS.	5.72470817E-02	-3.29087541E-01	-3.68027726E-01	-7.17131334E-02	-4.08055435E-03									
13	X-TRANS.	-1.41245136E-04	4.12876332E-03	-1.68176293E-03	-4.72652042E-03	-6.84278999E-03									
13	Y-TRANS.	2.45399222E-01	1.11059340E-01	5.02325428E-01	3.16201260E-02	-7.78259874E-02									
13	Z-TRANS.	-4.24708171E-01	-5.19221952E-01	-6.86190634E-01	-1.43644744E-01	2.441792984E-02									
14	X-TRANS.	-1.13299572E-04	3.31032886E-03	1.22229892E-03	-5.30397538E-03	-7.50541360E-03									
14	Y-TRANS.	1.53501946E-01	1.03238500E-01	4.61443449E-01	-8.44241498E-03	-5.82070160E-02									
14	Z-TRANS.	-2.25515564E-01	-4.477972719E-01	-6.30515758E-01	-6.99697489E-02	-6.14984842E-03									
15	X-TRANS.	1.79591440E-03	-1.58591941E-02	9.21717522E-03	-1.57745907E-02	-1.86487657E-02									
15	Y-TRANS.	1.85019455E-01	8.99027327E-01	3.56353244E-01	-6.38023822E-01	1.96491988E-01									
15	Z-TRANS.	3.75700394E-01	-3.71375637E-01	-4.18024081E-01	9.95694987E-01	-4.69034214E-01									
16	X-TRANS.	-1.55447471E-01	-3.01157943E-02	-5.84868727E-02	-8.59405731E-02	2.94716327E-02									
16	Y-TRANS.	-4.9525292E-01	-1.98100973E-01	2.40549834E-02	-2.02368779E-02	-1.92204417E-02									
16	Z-TRANS.	-9.5700891E-01	-6.99334692E-01	-9.57101972E-01	-1.79074681E-01	5.97229237E-02									
17	X-TRANS.	-1.55933852E-01	-2.99914729E-02	-5.80408587E-02	-8.85280079E-02	3.04330879E-02									
17	Y-TRANS.	-3.88132296E-01	-1.80361278E-01	1.84914968E-02	-2.85971427E-03	-5.85101778E-02									
17	Z-TRANS.	-5.46749983E-01	-6.42427050E-01	-8.84723679E-01	-6.68077149E-02	-6.57360546E-03									
18	X-TRANS.	2.15661479E-06	-5.73876795E-07	-5.85860327E-02	-9.03905844E-02	2.24426159E-04									
18	Y-TRANS.	-4.98638132E-01	-2.00231593E-01	-3.82976947E-06	-9.37610551E-05	-2.15461235E-02									
18	Z-TRANS.	2.38326844E-06	-6.61980900E-06	2.96924994E-01	-1.37816007E-02	3.56398047E-05									
19	X-TRANS.	2.15272374E-06	-5.85456230E-07	-5.79281858E-02	-9.05886917E-02	2.24315938E-04									
19	Y-TRANS.	-4.17023346E-01	-1.96521538E-01	-3.32514242E-06	-2.25986877E-04	-5.00216544E-02									
19	Z-TRANS.	-2.02919238E-06	-4.61279370E-06	3.21403990E-01	3.80674612E-02	-7.55738415E-05									
20	X-TRANS.	2.18190651E-06	-6.00741984E-07	-5.81319913E-02	-9.03760179E-02	2.24426159E-04									
20	Y-TRANS.	-3.9983264E-01	-1.81472994E-01	-3.14431007E-06	-2.60621126E-04	-5.73941490E-02									
20	Z-TRANS.	-3.37997743E-06	-2.67021769E-06	2.58411405E-01	3.96860012E-02	-1.00322867E-04									
21	X-TRANS.	1.55447471E-01	3.01157943E-02	-5.84845136E-02	-8.62211075E-02	-2.90385448E-02									
21	Y-TRANS.	-4.95525292E-01	-1.98054659E-01	-2.40625495E-02	2.80695544E-02	-1.93243829E-02									
21	Z-TRANS.	9.57003991E-01	6.99334692E-01	-9.57078269E-01	-1.79626173E-01	-5.88133391E-02									
22	X-TRANS.	1.55933852E-01	2.99914729E-02	-5.80408587E-02	-8.85280079E-02	3.04330879E-02									
22	Y-TRANS.	-3.88132296E-01	-1.80361278E-01	1.84914968E-02	-2.85971427E-03	-5.85101778E-02									
22	Z-TRANS.	5.46749983E-01	6.42427050E-01	-8.84702136E-01	-6.68077149E-02	6.57360546E-03									
23	X-TRANS.	1.451537696E-04	-4.13385827E-03	-1.68038201E-03	-4.67521381E-03	6.86877436E-03									
23	Y-TRANS.	2.53501946E-01	1.11116259E-01	-5.0232190E-01	-3.13447850E-02	-7.76959723E-02									
23	Z-TRANS.	4.24708171E-01	5.19221952E-01	-6.86173203E-01	-1.43660149E-01	-2.34560416E-02									
24	X-TRANS.	1.23540836E-04	-3.31542381E-03	-1.28135597E-03	-5.24381560E-03	7.53572975E-03									
24	Y-TRANS.	1.53501946E-01	1.03238500E-01	-4.61443449E-01	7.51228249E-03	-5.82503248E-02									
24	Z-TRANS.	2.25515564E-01	4.477972719E-01	-6.30499676E-01	-6.99132120E-02	6.48765699E-03									
25	X-TRANS.	-1.37159533E-03	1.58499305E-02	9.21360491E-03	-1.55675458E-02	1.87786921E-02									

CORRESPONDING MODE SHAPES FOR COMPLETE SYSTEM

NODAL POINT NUMBER	ELEMENT REFERENCE	6		7		8		9		10	
25	Y-TRANS.	-1.45019455E-01	9.00990037E-03	-3.56349144E-01	6.39774711E-01	1.93297137E-01					1.93297137E-01
25	Z-TRANS.	-3.35700399E-01	3.71421955E-01	-4.18012446E-01	1.00000000E+00	4.64270247E-01					4.64270247E-01
26	X-TRANS.	-1.56058366E-01	-3.70912459E-02	5.57178102E-02	7.81922355E-02	4.07232568E-02					4.07232568E-02
26	Y-TRANS.	1.00000000E+00	3.76653339E-01	-9.59388441E-01	-1.31694716E-01	-9.44521438E-02					-9.44521438E-02
26	Z-TRANS.	-6.25291829E-03	3.60676239E-01	-3.97216669E-01	-9.08826046E-02	-2.25534405E-02					-2.25534405E-02
27	X-TRANS.	-1.55447471E-01	-3.67577592E-02	5.53835922E-02	8.06288004E-02	4.18382633E-02					4.18382633E-02
27	Y-TRANS.	6.74455253E-01	3.55859194E-01	-9.20552490E-01	1.14199909E-02	-5.49584566E-02					-5.49584566E-02
27	Z-TRANS.	-5.72568033E-02	3.29133359E-01	-3.66601665E-01	-7.16743190E-02	4.42915951E-03					4.42915951E-03
28	X-TRANS.	1.46692607E-03	-5.62760537E-02	2.95267844E-02	4.57790606E-02	5.81203994E-02					5.81203994E-02
28	Y-TRANS.	2.54045603E-01	-2.83001390E-01	4.35373765E-02	6.81408877E-03	-1.31784653E-02					-1.31784653E-02
28	Z-TRANS.	-4.39994258E-01	-4.37841593E-02	2.21072733E-01	-2.55229386E-02	5.19272412E-03					5.19272412E-03
29	X-TRANS.	1.46692607E-03	-5.56739231E-02	2.91956148E-02	4.58561990E-02	5.82070160E-02					5.82070160E-02
29	Y-TRANS.	2.13424125E-01	-2.93749421E-01	5.99574374E-02	4.24594784E-02	-3.92593998E-03					-3.92593998E-03
29	Z-TRANS.	-3.68190661E-01	-6.35479399E-02	2.16768914E-01	-4.36534810E-02	3.15634474E-02					3.15634474E-02
30	X-TRANS.	1.46494054E-03	-5.59531941E-02	2.92994887E-02	4.57671396E-02	5.81203984E-02					5.81203984E-02
30	Y-TRANS.	2.00000000E-01	-2.49143122E-01	3.48385496E-02	5.09075206E-02	7.42312689E-03					7.42312689E-03
30	Z-TRANS.	-3.44455253E-01	-3.86521539E-02	1.97228862E-01	-4.88707766E-02	3.78822001E-02					3.78822001E-02
31	X-TRANS.	-9.22568093E-06	4.64566929E-06	2.84048208E-04	7.57794193E-04	-1.00563014E-05					-1.00563014E-05
31	Y-TRANS.	1.78501946E-04	-3.49884205E-01	-5.76881566E-06	-3.95901430E-05	-9.69207449E-03					-9.69207449E-03
31	Z-TRANS.	2.29474798E-05	-7.22556739E-06	3.36908208E-01	-1.16737598E-02	2.99313392E-05					2.99313392E-05
32	X-TRANS.	-3.22276265E-06	4.64566929E-06	2.83846018E-04	7.56797938E-04	-1.00476397E-05					-1.00476397E-05
32	Y-TRANS.	3.74704171E-04	-3.9133951E-01	-6.45620309E-06	1.11323679E-04	2.43971806E-02					2.43971806E-02
32	Z-TRANS.	-1.01342412E-06	-5.51644280E-06	3.65463079E-01	3.290866297E-02	-9.18536163E-05					-9.18536163E-05
33	X-TRANS.	-2.22665370E-06	4.64566929E-06	2.84039778E-04	7.57842391E-04	-1.00563014E-05					-1.00563014E-05
33	Y-TRANS.	5.21592607E-04	-3.63131079E-01	-6.06827729E-06	2.58600763E-05	5.54352534E-03					5.54352534E-03
33	Z-TRANS.	4.82008491E-07	-6.48449356E-05	3.48896117E-01	7.51246695E-03	-1.81333914E-05					-1.81333914E-05

GENERALIZED INERTIA PROPERTIES

VARIABLES	11	12	13	14	15
MX	1.31532959E+04	1.36310689E+04	8.20837363E+03	1.38481395E+04	1.10906660E+04
MY	6.61479808E+03	1.27906844E+04	1.00846576E+04	9.12324109E+03	1.24273351E+05
MZ	6.69924065E+03	1.26940061E+04	5.38717422E+03	1.69286570E+04	1.24311118E+05
PX	-4.99032160E+00	-2.17158100E+05	-1.84386166E+03	-3.75078907E+01	3.65542814E+05
PY	1.32640724E+01	8.45696133E+05	-1.07233413E+05	-1.04437073E+03	-8.44577469E+05
PZ	3.74334725E+01	8.48944696E+05	9.53436229E+04	8.58899507E+02	-8.45047303E+05
GM DIAGONAL	1.32336673E+04	1.95582795E+04	1.18401027E+04	1.99500187E+04	1.29837578E+05

NATURAL FREQUENCIES (RAD/SEC)

11	12	13	14	15
1.0749277E+03	1.05963809E+03	1.20336962E+03	1.2033375E+03	1.37674031E+03

CORRESPONDING MODE SHAPES FOR COMPLETE SYSTEM

MODAL POINT NUMBER	ELEMENT REFERENCE	11	12	13	14	15
1	X-TRANS.	3.61389446E-02	-7.00685258E-03	2.05755578E-01	-7.69559033E-02	-6.93589744E-01
1	Y-TRANS.	-1.06791360E-01	9.16776860E-03	7.462259152E-03	6.22569938E-02	-1.04743590E-02
1	Z-TRANS.	8.20086614E-02	6.10377805E-02	-1.39452333E-01	1.07292356E-01	-3.01709402E-02
2	X-TRANS.	3.04101536E-02	-9.41951775E-03	2.05755578E-01	-7.75248933E-02	-7.05555556E-01
2	Y-TRANS.	-9.49999005E-02	-1.35767099E-03	-2.14629817E-03	-1.32227264E-01	-7.69658120E-04
2	Z-TRANS.	1.81340459E-01	-1.52475824E-02	-1.70765720E-01	3.368422105E-02	-3.878663248E-02
3	X-TRANS.	-6.33711069E-06	-1.03901379E-03	2.40745436E-01	2.22201049E-03	-7.94444444E-01
3	Y-TRANS.	-1.02516143E-01	2.68700578E-06	-2.91455375E-04	6.19250030E-02	-1.67735043E-04
3	Z-TRANS.	3.36677355E-05	5.42301287E-01	4.59305274E-01	4.222664770E-03	-5.01709402E-01
4	X-TRANS.	-7.55208354E-06	-2.16883592E-02	2.14376258E-01	1.90624941E-03	-7.70632479E-01
4	Y-TRANS.	-9.58345135E-02	7.35130874E-06	6.95360041E-04	-1.32953106E-01	1.11581197E-04
4	Z-TRANS.	-5.9786239E-06	-7.03963551E-02	-2.52535497E-01	-2.32354007E-03	-1.38162393E-01
5	X-TRANS.	-5.02115342E-05	-6.99552894E-01	-5.70366556E-01	-5.31331532E-03	-4.1440179E-01
5	Y-TRANS.	1.08000000E+00	-3.3243403E-05	-5.7860046E-04	5.7714557E-02	7.7931624E-05
5	Z-TRANS.	5.23914436E-05	1.00000000E+00	1.00000000E+00	9.12015552E-03	-7.606883761E-01
6	X-TRANS.	-3.61389446E-02	-7.00685258E-03	2.04868154E-01	8.07491702E-02	-6.93589744E-01
6	Y-TRANS.	-1.06791360E-01	9.16219826E-03	-8.06795132E-03	6.21147463E-02	1.01367521E-01
6	Z-TRANS.	-8.20086614E-02	6.10688828E-02	-1.37423935E-01	-1.09853011E-01	-3.78846154E-02
7	X-TRANS.	-3.84324204E-02	-9.52217635E-03	2.04984929E-01	8.13181603E-02	-7.05982906E-01
7	Y-TRANS.	-9.49699800E-02	1.37237678E-03	3.54208925E-03	-1.32479048E-01	9.94017094E-04
7	Z-TRANS.	-1.01340459E-01	-1.52176914E-02	-1.70385396E-01	-3.68326221E-02	-3.89529915E-02
8	X-TRANS.	-8.15408595E-05	2.02148108E-02	3.29107202E-02	-7.42057847E-02	-5.01282051E-01
8	Y-TRANS.	-0.64217324E-02	2.87517310E-03	-7.12221095E-02	-7.00866761E-02	-1.86282051E-01
8	Z-TRANS.	-1.14740592E-01	-1.09059794E-03	-1.35268763E-01	-9.46556097E-02	1.03632479E-01
9	X-TRANS.	-8.24092630E-05	2.03143778E-02	3.29741379E-02	-7.43480322E-02	-5.01282051E-01
9	Y-TRANS.	-9.47227789E-02	1.13390395E-02	-0.32657201E-02	-3.29397019E-02	1.74017094E-01
9	Z-TRANS.	-1.63816522E-01	-6.10411602E-03	-1.19523327E-01	-1.111948791E-01	-1.01196581E-01
10	X-TRANS.	-9.18303273E-05	1.988105162E-02	3.28600406E-02	-7.40635372E-02	-5.20089470E-01
10	Y-TRANS.	-1.01024271E-01	2.26428063E-02	-7.80793103E-02	-1.94642010E-02	4.39743590E-01
10	Z-TRANS.	-1.74972167E-01	-1.25801826E-02	-1.03803245E-01	-1.05832148E-01	-2.53584274E-01
11	X-TRANS.	3.59385138E-02	-8.55836238E-03	-1.54792089E-01	-1.93009199E-01	-7.04700855E-01
11	Y-TRANS.	-1.73836552E-02	-5.03561002E-02	-1.85891927E-01	-3.79943010E-02	2.47094017E-02
11	Z-TRANS.	-1.34001336E-01	-2.26245723E-02	-6.31465517E-02	-6.82313893E-02	1.02136752E-02
12	X-TRANS.	3.02097528E-02	-1.09870969E-02	-1.55172414E-01	-1.93172119E-01	-7.16239316E-01

CORRESPONDING MODE SHAPE FOR COMPLETE SYSTEM

NODAL POINT NUMBER	ELEMENT REFERENCE	11					13					14					15				
		12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	
12	Y-TRANS.	-1.07370296E-01	1.46042893E-02	-4.62347870E-02	-1.68335704E-01	3.96153866E-02															
12	Z-TRANS.	-1.74192830E-01	7.07418857E-03	-1.28803245E-01	-1.06221147E-02	1.60940171E-02															
13	X-TRANS.	1.14360993E-03	-3.230111870E-03	-1.17761156E-01	-2.71977240E-01	-7.97963248E-01															
13	Y-TRANS.	4.81407259E-02	-4.67521340E-01	1.7834313E-01	4.5677403E-01	4.35897436E-01															
13	Z-TRANS.	-9.13392320E-02	-2.69383109E-01	1.50228195E-01	2.53475581E-01	2.599444444E-01															
14	X-TRANS.	1.04141617E-03	-2.28994163E-02	-1.04627282E-01	-2.411963016E-01	-7.81196581E-01															
14	Y-TRANS.	4.76951908E-02	6.09381153E-02	-6.20416430E-02	-2.75296349E-01	1.32136752E-02															
14	Z-TRANS.	-3.46804721E-02	3.53851146E-02	-1.39072008E-01	-9.51114272E-02	1.98418803E-03															
15	X-TRANS.	-2.11957248E-03	-6.98073290E-01	2.87145030E-01	5.52441916E-01	-4.06025641E-01															
15	Y-TRANS.	-5.05010020E-01	-8.59692968E-01	3.97437148E-01	1.0000000E+00	6.71367521E-01															
15	Z-TRANS.	9.61055444E-01	-5.01926913E-01	3.02104462E-01	5.31531532E-01	3.86111111E-01															
16	X-TRANS.	-3.4001330E-02	-7.89860927E-03	-4.65770731E-02	-2.72024656E-01	-6.94017094E-01															
16	Y-TRANS.	1.23135159E-01	-4.86591729E-02	2.57805491E-02	-2.01327643E-01	2.22592308E-02															
16	Z-TRANS.	-5.29949474E-02	-3.79802994E-02	6.7662761E-02	-1.42437174E-01	3.40940171E-02															
17	X-TRANS.	-2.62725451E-02	-1.03025551E-02	-4.61713996E-02	-2.72403983E-01	-7.05982906E-01															
17	Y-TRANS.	2.02560677E-01	1.28782593E-02	-9.22160243E-03	-2.06495970E-01	3.29358974E-01															
17	Z-TRANS.	4.77176575E-03	9.17938117E-03	-1.10509135E-01	-4.86960645E-02	1.44273504E-02															
18	X-TRANS.	4.28412340E-06	2.01194091E-02	-6.44903651E-02	-6.07396871E-04	-5.01282051E-01															
18	Y-TRANS.	1.31462926E-01	-1.27625874E-05	1.24987223E-03	-2.28022760E-01	1.93974359E-05															
18	Z-TRANS.	1.34224004E-06	2.19826879E-03	-1.12145030E-02	-1.08345187E-04	-2.14444444E-01															
19	X-TRANS.	4.30416398E-06	2.02243393E-02	-6.4629174E-02	-6.08345187E-04	-5.01282051E-01															
19	Y-TRANS.	1.4775551E-01	-2.02600972E-05	1.1814431E-03	-2.1507236E-01	1.67222222E-05															
19	Z-TRANS.	-5.08550434E-07	1.28509142E-02	2.39224138E-02	2.25462304E-04	2.02094017E-01															
20	X-TRANS.	4.27521710E-06	1.89144732E-02	-6.43128803E-02	-6.05974395E-04	-5.19658120E-01															
20	Y-TRANS.	2.00299468E-01	-2.18969178E-05	1.0489509E-03	-1.90753912E-01	1.41965812E-05															
20	Z-TRANS.	-1.00956669E-06	2.58707462E-02	3.09964503E-02	2.95448090E-04	5.0792308E-01															
21	X-TRANS.	3.40236028E-02	-7.90704244E-03	-4.95436105E-02	2.7112755E-01	-6.94017094E-01															
21	Y-TRANS.	1.23135159E-01	4.65257905E-02	-2.35420892E-02	-2.01801802E-01	-2.52350427E-02															
21	Z-TRANS.	5.29949474E-02	-3.79834224E-02	6.62905680E-02	1.43669986E-01	3.40726496E-02															
22	X-TRANS.	3.62725451E-02	-1.03113189E-02	-4.91506089E-02	2.71503082E-01	-7.05982906E-01															
22	Y-TRANS.	2.02560677E-01	-1.29148134E-02	1.14903722E-02	-2.06306306E-01	3.29059829E-02															
22	Z-TRANS.	-4.76951908E-03	9.17992402E-03	-1.11029412E-01	4.66287340E-02	1.44189034E-02															
23	X-TRANS.	-1.14117123E-03	-3.23172245E-03	-1.20727688E-01	2.6978696E-01	-7.98290598E-01															
23	Y-TRANS.	4.81957594E-02	4.67508497E-01	-1.81414807E-01	4.63442390E-01	4.35997436E-01															
23	Z-TRANS.	9.12936945E-02	-2.69400992E-01	1.52763692E-01	-2.30678046E-01	-2.59487179E-01															
24	X-TRANS.	-1.03719548E-03	-2.28970946E-02	-1.07264199E-01	2.39971550E-01	-7.81196581E-01															
24	Y-TRANS.	4.76951908E-02	-6.0957550E-02	6.50988444E-02	-2.74110953E-01	-1.31965812E-02															
24	Z-TRANS.	3.47072738E-02	3.533700628E-02	-1.39959432E-01	8.25509720E-02	1.98247863E-03															
25	X-TRANS.	2.02961479E-03	-6.19807528E-01	2.94244422E-01	-6.47226174E-01	-4.05940171E-01															

GENERALIZED INERTIA PROPERTIES

VARIABLES	16	17	18	19	20
MNX	4.496377947E+04	4.41563235E+04	1.37547312E+04	6.83417642E+04	1.44983817E+04
MNY	4.664761147E+04	1.70675464E+04	2.17145598E+04	4.25923677E+04	1.13153543E+04
MNZ	1.81974894E+04	4.61491339E+04	2.11874296E+04	4.46596751E+04	1.89149873E+04
PX	4.23751146E+03	6.36749431E+00	4.01127806E+05	2.68242553E+02	-4.51132124E+01
PY	1.34490755E+05	1.46051499E+02	1.47309299E+06	5.01191735E+02	3.30674779E+02
PZ	-1.35196947E+05	-4.29227309E+02	1.48464538E+06	4.73750979E+02	-6.62234658E+02
GM DIAGONAL	5.48594422E+04	5.36365059E+04	3.08284678E+04	7.77964035E+04	2.23643617E+04

NATURAL FREQUENCIES (RAD/SEC)

16	17	18	19	20
2.46526099E+03	2.49077254E+03	2.6732300E+03	2.69850389E+03	2.81151036E+03

CORRESPONDING MODE SHAPES FOR COMPLETE SYSTEM

16	17	18	19	20			
1	X-TRANS.	-1.34243117E-01	3.64965145E-01	-1.37659033E-01	3.96046852E-01	3.96046852E-01	-4.20041831E-03
1	Y-TRANS.	-4.94044685E-03	1.00000000E+00	3.17981340E-02	3.49973598E-01	3.49973598E-01	8.33442845E-02
1	Z-TRANS.	3.42547550E-01	3.50798231E-01	1.65055131E-01	-3.36017570E-01	-3.36017570E-01	-1.87707338E-01
2	X-TRANS.	-1.599467659E-01	3.69237638E-01	-1.31906165E-01	4.05807711E-01	4.05807711E-01	-6.00741363E-03
2	Y-TRANS.	-5.91943755E-03	-6.70339595E-01	3.10347752E-02	-7.28992143E-01	-7.28992143E-01	-1.26498103E-01
2	Z-TRANS.	-2.12920513E-01	-3.78007646E-01	3.59796438E-01	2.56174719E-01	2.56174719E-01	2.57593388E-01
3	X-TRANS.	-1.54267900E-01	-3.95997382E-04	-9.05952417E-02	-2.11298194E-04	-2.11298194E-04	-1.29207063E-03
3	Y-TRANS.	-1.92461159E-03	9.91679794E-01	-7.7175725E-05	3.41922889E-01	3.41922889E-01	3.24469595E-02
3	Z-TRANS.	-1.74024122E-01	-5.05959531E-04	8.7446390E-01	3.34797462E-06	3.34797462E-06	-3.79430632E-03
4	X-TRANS.	-1.45574655E-01	-3.73295731E-04	-3.43002545E-02	-2.40073087E-04	-2.40073087E-04	-1.59667883E-03
4	Y-TRANS.	1.22001654E-03	-6.55273214E-01	6.61238738E-05	-7.16203026E-01	-7.16203026E-01	-1.26909561E-01
4	Z-TRANS.	-2.84076334E-01	-6.12772656E-04	8.49872774E-01	2.77208394E-04	2.77208394E-04	8.96870468E-04
5	X-TRANS.	-5.69561621E-02	-3.62716438E-04	3.52303242E-01	-1.04294778E-04	-1.04294778E-04	-5.75094300E-03
5	Y-TRANS.	1.26964433E-04	-6.79109512E-02	1.4003927E-05	-1.40287945E-01	-1.40287945E-01	-3.29607951E-02
5	Z-TRANS.	-3.01157982E-01	-6.35484596E-04	2.08921035E-01	-6.11517814E-05	-6.11517814E-05	-5.75172124E-04
6	X-TRANS.	-1.72919789E-01	-3.65414896E-01	-1.37574215E-01	-3.96046852E-01	-3.96046852E-01	4.75880806E-03
6	Y-TRANS.	1.24974322E-03	1.00000000E+00	-3.19509058E-02	3.49973598E-01	3.49973598E-01	8.36901363E-02
6	Z-TRANS.	3.43370968E-01	-3.49449037E-01	1.64970314E-01	3.36017570E-01	3.36017570E-01	1.82706920E-01
7	X-TRANS.	-1.78544251E-01	-3.69312300E-01	-3.31721798E-01	-4.06051733E-01	-4.06051733E-01	6.27625042E-03
7	Y-TRANS.	9.01406121E-03	-6.70339595E-01	-3.08990570E-02	-7.28892143E-01	-7.28892143E-01	-1.26666701E-01
7	Z-TRANS.	-2.14143921E-01	3.777198163E-01	3.59881255E-01	-2.57694667E-01	-2.57694667E-01	-2.53123921E-01
8	X-TRANS.	3.40529363E-01	-5.84666381E-01	-1.87953495E-01	-2.41633824E-03	-2.41633824E-03	1.13337933E-04
8	Y-TRANS.	2.75765095E-01	-3.59680634E-02	-1.69720102E-02	3.63936018E-01	3.63936018E-01	1.64550163E-01
8	Z-TRANS.	2.23904053E-01	2.83112210E-01	-2.43256697E-02	6.32991703E-01	6.32991703E-01	1.45054896E-01
9	X-TRANS.	3.42349048E-01	-5.87812008E-01	-1.88543618E-01	-2.09614446E-03	-2.09614446E-03	1.13420440E-01
9	Y-TRANS.	-1.38213400E-01	2.10029233E-01	6.24113933E-02	-1.45534407E-01	-1.45534407E-01	-1.53409617E-01
9	Z-TRANS.	5.46476437E-02	-1.42509562E-01	-3.70399435E-02	-2.56222548E-01	-2.56222548E-01	-8.6226283E-02
10	X-TRANS.	3.33995037E-01	-5.73620233E-01	-1.89053524E-01	-1.97219155E-03	-1.97219155E-03	1.07443263E-01
10	Y-TRANS.	-2.16454834E-01	2.45109052E-01	1.51223656E-01	-2.51930161E-01	-2.51930161E-01	-2.17452491E-01
10	Z-TRANS.	-5.33250620E-04	-2.23903735E-01	-8.29262037E-02	-4.44363104E-01	-4.44363104E-01	-1.42176237E-01
11	X-TRANS.	3.92066170E-01	-5.922934034E-02	-1.32230704E-01	3.91994937E-01	3.91994937E-01	5.61630399E-02
11	Y-TRANS.	-5.40612076E-01	6.49903306E-01	-2.15012723E-01	-2.02757443E-01	-2.02757443E-01	3.17480244E-01
11	Z-TRANS.	3.89926703E-01	-1.59433366E-01	-9.33842239E-02	1.00000000E+00	1.00000000E+00	1.29362204E-01
12	X-TRANS.	3.96331344E-01	-5.69822322E-02	-1.24427491E-01	4.02635432E-01	4.02635432E-01	3.29974573E-02

CORRESPONDING NODE SHAPES FOR COMPLETE SYSTEM

16	17	18	19	20	
12	Y-TRANS.	5.00413565E-01	-3.08142494E-01	1.76891156E-01	-2.66639904E-01
12	Z-TRANS.	-3.87675765E-01	1.00382280E-01	-1.45122986E-01	-8.44618099E-02
13	X-TRANS.	9.52026468E-02	1.35956825E-01	-6.39525021E-02	1.75988287E-02
13	Y-TRANS.	-5.14371050E-01	1.18641731E-01	-7.64634043E-01	-5.00244021E-01
13	Z-TRANS.	6.93879239E-01	-4.87169935E-01	-4.49109415E-01	8.17471930E-01
14	X-TRANS.	8.57733664E-02	1.18956600E-01	-5.32739310E-03	1.96046892E-02
14	Y-TRANS.	1.65922250E-01	-3.75759399E-01	-6.85495183E-01	3.90678309E-01
14	Z-TRANS.	-5.66455500E-01	1.50646278E-01	-3.96163206E-01	-6.31771596E-01
15	X-TRANS.	4.76923077E-02	3.42478075E-02	1.00000000E+00	4.70473402E-02
15	Y-TRANS.	-1.07692308E-01	-2.36587812E-01	-1.78803189E-01	7.8863241E-02
15	Z-TRANS.	-1.29693962E-01	-1.01506634E-01	-9.78799589E-02	-1.26226711E-01
16	X-TRANS.	-2.31134422E-01	2.90532944E-01	-1.32145886E-01	3.90678309E-01
16	Y-TRANS.	4.50206742E-03	3.81380706E-01	-1.54792197E-01	-8.16251830E-01
16	Z-TRANS.	1.00000000E+00	-3.42478075E-01	-1.35366957E-01	6.42996584E-01
17	X-TRANS.	-2.31830935E-01	2.97054194E-01	-1.24427481E-01	3.98487067E-01
17	Y-TRANS.	-9.66087676E-02	-1.69732404E-01	-2.78625954E-01	6.30551489E-01
17	Z-TRANS.	-7.39867659E-01	2.79739150E-01	-1.94147563E-01	-5.08540752E-01
18	X-TRANS.	-6.80810587E-01	-1.17247593E-03	-1.92281595E-01	-1.33504148E-04
18	Y-TRANS.	-6.69148056E-04	3.83954284E-01	1.37404580E-04	-7.80624699E-01
18	Z-TRANS.	-2.54673294E-01	-4.41421193E-04	-5.91772899E-03	-2.01854563E-05
19	X-TRANS.	-6.85277089E-01	-1.18034630E-03	-1.93129771E-01	-1.34187400E-04
19	Y-TRANS.	7.83788255E-05	3.91429888E-02	-4.14503617E-05	3.24792582E-01
19	Z-TRANS.	3.02594103E-01	5.17652350E-01	7.76679146E-02	5.49553639E-05
20	X-TRANS.	-6.67080232E-01	-1.14308927E-03	-1.92875318E-01	-1.32910497E-04
20	Y-TRANS.	2.56079404E-04	-1.40274342E-01	-7.98218830E-05	5.52952660E-01
20	Z-TRANS.	3.86435070E-01	6.59545761E-04	1.78456319E-01	1.10566130E-04
21	X-TRANS.	-2.30355666E-01	-2.91432826E-01	-1.32315522E-01	3.90678309E-01
21	Y-TRANS.	-5.83540116E-03	3.81380706E-01	1.55046550E-01	-9.162251830E-01
21	Z-TRANS.	9.99172870E-01	3.46076006E-01	-1.35114504E-01	-6.42996584E-01
22	X-TRANS.	-2.30355666E-01	-2.97728806E-01	-1.24597116E-01	3.98233045E-01
22	Y-TRANS.	9.7187533E-02	-1.69440072E-01	2.78455319E-01	6.80795510E-01
22	Z-TRANS.	-7.38875103E-01	-2.82437598E-01	-1.94317218E-01	5.08296730E-01
23	X-TRANS.	9.56989247E-02	-1.35597032E-01	-6.39694566E-02	-1.76134700E-02
23	Y-TRANS.	5.1447473E-01	1.20283337E-01	7.64800679E-01	-4.99511957E-01
23	Z-TRANS.	6.92307692E-01	4.89543512E-01	-4.48854962E-01	-8.17715995E-01
24	X-TRANS.	8.61869313E-02	-1.18664268E-01	-5.34690416E-03	1.95510095E-02
24	Y-TRANS.	-1.64598842E-01	-3.76533551E-01	6.85325448E-01	3.9116642E-01
24	Z-TRANS.	-5.67659222E-01	-1.60535192E-01	-3.96352641E-01	6.31527574E-01
25	X-TRANS.	4.76923077E-02	-3.43152687E-02	1.00000000E+00	-4.59980476E-02
					1.00000000E+00

CORRESPONDING MODE SHAPES FOR COMPLETE SYSTEM

MODAL POINT NUMBER	ELEMENT REFERENCE	16	17	18	19	20
25	Y-TRANS.	1.08519438E-01	-2.39133071E-01	1.74724243E-01	7.20351391E-02	8.26397208E-02
25	Z-TRANS.	-1.30024814E-01	1.01056492E-01	-9.78795549E-02	1.26159102E-01	-6.34806169E-02
26	X-TRANS.	3.79900744E-01	6.06701147E-02	-1.32144589E-01	-3.92142509E-01	5.62708952E-02
26	Y-TRANS.	5.38130697E-01	5.91702271E-01	2.15182758E-01	-2.02513621E-01	3.20647781E-01
26	Z-TRANS.	6.99330025E-01	1.61929331E-01	-9.30449594E-02	-1.00000000E+00	-1.31776111E-01
27	X-TRANS.	3.46765922E-01	5.93764336E-02	-1.24342663E-01	-4.02979453E-01	3.29447725E-02
27	Y-TRANS.	-4.98593879E-01	-5.06183944E-01	3.07972856E-01	1.76915569E-01	-2.70017209E-01
27	Z-TRANS.	-3.47344913E-01	-1.01709017E-01	-1.45377439E-01	7.73060029E-01	8.74784442E-02
28	X-TRANS.	3.78461538E-01	5.95788172E-01	-1.87971077E-01	1.94460713E-03	-1.12707562E-01
28	Y-TRANS.	-2.75434243E-01	-9.69574934E-02	1.68532655E-02	3.63936018E-01	1.65939992E-01
28	Z-TRANS.	2.24996609E-01	-2.82212724E-01	-2.42409621E-02	-6.32991703E-01	-1.48249785E-01
29	X-TRANS.	3.40291222E-01	5.88936362E-01	-1.88464801E-01	1.92020494E-03	-1.12739901E-01
29	Y-TRANS.	1.37464993E-01	2.10501462E-01	-6.20186192E-02	-1.42532018E-01	-1.25212732E-01
29	Z-TRANS.	3.31265909E-02	1.42702946E-01	-3.70659306E-02	2.56446569E-01	9.86485577E-02
30	X-TRANS.	3.32009925E-01	5.74544637E-01	-1.88888889E-01	1.79331381E-03	-1.06795661E-01
30	Y-TRANS.	2.15632754E-01	2.46008545E-01	-1.51060221E-01	-2.518330161E-01	-2.18565039E-01
30	Z-TRANS.	-1.44592299E-03	2.23891259E-01	-8.29886175E-02	4.44353104E-01	1.25212927E-01
31	X-TRANS.	-4.21936228E-04	1.55947930E-06	3.49957591E-02	1.89507077E-05	-6.56652820E-06
31	Y-TRANS.	6.17535153E-04	-3.54396222E-01	-4.89652248E-05	-2.23230844E-03	1.19738962E-01
31	Z-TRANS.	-3.57999030E-01	-6.15695975E-04	-3.15097540E-03	-2.36603221E-05	-5.76414898E-04
32	X-TRANS.	-4.15467328E-04	1.50326053E-06	3.54198473E-02	1.90800390E-05	-6.68557687E-06
32	Y-TRANS.	-6.31430935E-04	3.62041826E-01	5.03338253E-05	1.89482874E-03	-1.19215442E-01
32	Z-TRANS.	3.63523573E-01	6.23791320E-04	2.52926209E-03	2.06295754E-05	5.69651449E-04
33	X-TRANS.	-4.20943672E-04	1.54171351E-06	3.53519932E-02	1.91020010E-05	-6.63086219E-06
33	Y-TRANS.	4.42944327E-05	-4.94821228E-02	-6.63189143E-06	-4.63884922E-04	1.78489465E-02
33	Z-TRANS.	-4.99338296E-02	-8.84174094E-05	-7.40033927E-04	-4.74621767E-06	-8.77396378E-05

CPU TIME USAGE TABLE (SEC)

	TIME IN	TIME OUT	TOTAL
INPUT AND INITIALIZATION OVERLAY	.300	1.229	.93
STRUCTURAL STIFFNESS MATRIX GENERATION OVERLAY	1.229	2.128	.90
NORMAL MODE ANALYSIS OVERLAY	2.128	116.605	114.46

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APPENDIX C

EXAMPLE OF INPUT AND OUTPUT

LANDING GEAR OPTION

STRUCTURAL ANALYSIS PROGRAM

COMPRESSION CRUSH DISTANCE CARDS
 10 2 8.36 11.35 14.15 27.94 1.E20
 TENSIGN CRUSH DISTANCE CARDS
 11 2 1.E20
 COMPRESSION SPRING RATE CARDS
 12 2 2482.E06 848.E06 1023.E06 1198.E06 2.98E10
 TENSIGN SPRING RATE CARDS
 13 2 2.82E10
 COMPRESSION PLASTIC FORCE CARDS
 14 2 1.07E09 1.54E09 1.92E09 3.34E09 1.E30
 TENSIGN PLASTIC FORCE CARDS
 15 2 1.E30
 DATA TERMINATOR CARD
 16

INITIAL CONDITIONS FOR GEAR ANALYSIS

COORDINATE DEFINITIONS			EXTERNAL FORCES			EXTERNAL MOMENTS			
MODAL POINT	X	Y	Z	X	Y	Z	X	Y	Z
1	0.	2.5500E+01	0.	0.	0.	0.	0.	0.	0.
2	6.0000E+01	0.	1.95600E+01	0.	0.	0.	0.	0.	0.
3	0.	-2.3500E+01	0.	0.	0.	0.	0.	0.	0.
4	-2.5150E+01	0.	3.0000E+01	0.	0.	0.	0.	0.	0.

STRUCTURAL ELEMENT	AXIAL FORCE			AXIAL STIFFNESS			SHEAR FORCE			SHEAR MOMENT		
	X	Y	Z	X	Y	Z	X	Y	Z	X	Y	Z
1	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
2	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
3	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.

TOTAL ENERGY ABSORBED			NUMBER OF UNCRUSHED FOOTPAD CRUSH LEVELS = 2			
SURFACE COORDINATE SYSTEM	LANDER COORDINATE SYSTEM			FOOTPAD ATTENUATION ENERGY THIS STEP		
	X	Y	Z	X	Y	Z
0.	0.	0.	0.	0.	0.	0.

INCREMENTAL STEP 1
19 ITERATIONS REQUIRED FOR TOLERANCE SATISFACTION

NODAL POINT	COORDINATE DEFINITIONS			EXTERNAL FORCES			EXTERNAL MOMENTS		
	X	Y	Z	X	Y	Z	X	Y	Z
1	0.	2.35000E+01	0.	-8.02075E+07	-7.57100E+07	1.65051E+08	0.	0.	0.
2	6.08000E+01	0.	1.95800E+01	-4.55990E+08	0.	1.66227E+08	0.	0.	0.
3	0.	-2.35000E+01	0.	-8.02075E+07	7.57100E+07	1.64051E+08	0.	0.	0.
4	-2.48960E+01	0.	5.69206E+01	6.20405E+08	0.	-4.96529E+08	0.	0.	0.

STRUCTURAL ELEMENT	AXIAL FORCE		AXIAL STIFFNESS	SHEAR FORCE		SHEAR MOMENT	SHEAR FORCE		SHEAR MOMENT
	X	Y		X	Y		X	Y	
1	-1.97682E+08	-3.99649E-03	9.49697E+10	0.	0.	0.	0.	0.	0.
2	-4.89787E+08	-1.97336E-01	2.48200E+09	0.	0.	0.	0.	0.	0.
3	-1.97682E+08	-3.99645E-03	9.49657E+10	0.	0.	0.	0.	0.	0.

TOTAL ENERGY ABSORBED

SURFACE COORDINATE SYSTEM	LANDER COORDINATE SYSTEM			SURFACE COORDINATE SYSTEM			LANDER COORDINATE SYSTEM		
	X	Y	Z	X	Y	Z	X	Y	Z
7.879E+07	0.	-2.454E+07	7.879E+07	0.	-2.994E+07	0.	0.	0.	0.

INCREMENTAL STEP 2
20 ITERATIONS REQUIRED FOR TOLERANCE SATISFACTION

NODAL POINT	COORDINATE DEFINITIONS			EXTERNAL FORCES			EXTERNAL MOMENTS		
	X	Y	Z	X	Y	Z	X	Y	Z
1	0.	2.35000E+01	0.	-1.56136E+08	-1.48900E+08	3.23397E+08	0.	0.	0.
2	6.08000E+01	0.	1.95800E+01	-9.19949E+08	0.	3.58504E+08	0.	0.	0.
3	0.	-2.35000E+01	0.	-1.56136E+08	1.48900E+08	3.23397E+08	0.	0.	0.
4	-2.48960E+01	0.	5.10399E+01	1.23162E+09	0.	-9.82968E+08	0.	0.	0.

STRUCTURAL ELEMENT	AXIAL FORCE		AXIAL STIFFNESS	SHEAR FORCE		SHEAR MOMENT	SHEAR FORCE		SHEAR MOMENT
	X	Y		X	Y		X	Y	
1	-3.88761E+08	-7.07240E-03	9.49688E+10	0.	0.	0.	0.	0.	0.
2	-9.79684E+08	-3.94715E-01	2.48200E+09	0.	0.	0.	0.	0.	0.
3	-3.88761E+08	-7.07240E-03	9.49688E+10	0.	0.	0.	0.	0.	0.

TOTAL ENERGY ABSORBED

SURFACE COORDINATE SYSTEM	LANDER COORDINATE SYSTEM			SURFACE COORDINATE SYSTEM			LANDER COORDINATE SYSTEM		
	X	Y	Z	X	Y	Z	X	Y	Z
1.714E+09	0.	-1.163E+08	1.714E+09	0.	-1.183E+08	1.400E+09	0.	0.	1.400E+09

INCREMENTAL STEP 9
20 ITERATIONS REQUIRED FOR TOLERANCE SATISFACTION

MODAL POINT	COORDINATE DEFINITIONS			EXTERNAL FORCES			EXTERNAL MOMENTS		
	X	Y	Z	X	Y	Z	X	Y	Z
1	0.	2.35000E+01	0.	-1.40797E+08	-1.44713E+08	3.19368E+08	0.	0.	0.
2	6.08000E+01	0.	1.95600E+01	-9.96270E+08	0.	3.65173E+08	0.	0.	0.
3	0.	-2.35000E+01	0.	-1.40797E+08	1.44713E+08	-3.19368E+08	0.	0.	0.
4	-2.26400E+01	0.	5.18605E+01	1.27966E+09	0.	-1.02389E+09	0.	0.	0.

STRUCTURAL ELEMENT	AXIAL FORCE			AXIAL STIFFNESS			SHEAR FORCE			SHEAR MOMENT		
	X	Y	Z	X	Y	Z	X	Y	Z	X	Y	Z
1	-3.77931E+08	-6.07392E-03	9.49688E+10	0.	0.	0.	0.	0.	0.	0.	0.	0.
2	-1.07008E+09	-1.76881E+00	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
3	-3.77931E+08	-6.07392E-03	9.49688E+10	0.	0.	0.	0.	0.	0.	0.	0.	0.

TOTAL ENERGY ABSORBED
NUMBER OF UNCRUSHED FOOTPAD CRUSH LEVELS = 1
FOOTPAD ATTENUATION ENERGY THIS STEP

SURFACE COORDINATE SYSTEM	LANDER COORDINATE SYSTEM			SURFACE COORDINATE SYSTEM			LANDER COORDINATE SYSTEM		
	X	Y	Z	X	Y	Z	X	Y	Z
4.031E+09	0.	-9.746E+08	4.031E+09	0.	-9.746E+08	0.	0.	0.	0.

INCREMENTAL STEP 10
20 ITERATIONS REQUIRED FOR TOLERANCE SATISFACTION

MODAL POINT	COORDINATE DEFINITIONS			EXTERNAL FORCES			EXTERNAL MOMENTS		
	X	Y	Z	X	Y	Z	X	Y	Z
1	0.	2.35000E+01	0.	-1.36988E+08	-1.42380E+08	3.19368E+08	0.	0.	0.
2	6.08000E+01	0.	1.95600E+01	-9.97426E+08	0.	3.67351E+08	0.	0.	0.
3	0.	-2.35000E+01	0.	-1.36988E+08	1.42380E+08	-3.19368E+08	0.	0.	0.
4	-2.26400E+01	0.	5.19723E+01	1.27140E+09	0.	-1.01712E+09	0.	0.	0.

STRUCTURAL ELEMENT	AXIAL FORCE			AXIAL STIFFNESS			SHEAR FORCE			SHEAR MOMENT		
	X	Y	Z	X	Y	Z	X	Y	Z	X	Y	Z
1	-3.71749E+08	-6.76279E-03	9.49688E+10	0.	0.	0.	0.	0.	0.	0.	0.	0.
2	-1.07008E+09	-1.96548E+00	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
3	-3.71749E+08	-6.76279E-03	9.49688E+10	0.	0.	0.	0.	0.	0.	0.	0.	0.

TOTAL ENERGY ABSORBED
NUMBER OF UNCRUSHED FOOTPAD CRUSH LEVELS = 1
FOOTPAD ATTENUATION ENERGY THIS STEP

SURFACE COORDINATE SYSTEM	LANDER COORDINATE SYSTEM			SURFACE COORDINATE SYSTEM			LANDER COORDINATE SYSTEM		
	X	Y	Z	X	Y	Z	X	Y	Z
4.355E+09	0.	-1.068E+09	4.355E+09	0.	-1.068E+09	0.	0.	0.	0.

INCREMENTAL STEP 59
12 ITERATIONS REQUIRED FOR TOLERANCE SATISFACTION

NODAL POINT	COORDINATE DEFINITIONS			EXTERNAL FORCES			EXTERNAL MOMENTS				
	X	Y	Z	X	Y	Z	X	Y	Z		
1	0.	2.35000E+01	0.	-5.29597E+07	-1.22447E+08	2.90537E+08	0.	0.	0.		
2	6.08000E+01	0.	1.95800E+01	-1.71052E+09	0.	8.72077E+08	0.	0.	0.		
3	0.	-2.35000E+01	0.	-5.29597E+07	1.22447E+08	2.90537E+08	0.	0.	0.		
4	-1.01644E+01	0.	5.57597E+01	1.81644E+09	0.	-1.45315E+09	0.	0.	0.		
STRUCTURAL ELEMENT	AXIAL FORCE	AXIAL STROKE	AXIAL STIFFNESS	SHEAR FORCE	SHEAR MOMENT	SHEAR FORCE	SHEAR MOMENT	LANDER COORDINATE SYSTEM	LANDER COORDINATE SYSTEM		
1	-3.19702E+08	-5.81619E-03	5.49677E+10	0.	0.	0.	0.	NODE 4	0.		
2	-1.92000E+09	-1.17898E+01	0.	0.	0.	0.	0.	NODE 4	0.		
3	-3.19702E+08	-5.81619E-03	5.49677E+10	0.	0.	0.	0.	NODE 4	0.		
TOTAL ENERGY ABSORBED											
SURFACE COORDINATE SYSTEM			LANDER COORDINATE SYSTEM			SURFACE COORDINATE SYSTEM			LANDER COORDINATE SYSTEM		
X	Y	Z	X	Y	Z	X	Y	Z	X	Y	Z
2.023E+10	0.	-4.873E+09	2.023E+10	0.	-4.873E+09	0.	0.	0.	0.	0.	0.

INCREMENTAL STEP 60
12 ITERATIONS REQUIRED FOR TOLERANCE SATISFACTION

NODAL POINT	COORDINATE DEFINITIONS			EXTERNAL FORCES			EXTERNAL MOMENTS				
	X	Y	Z	X	Y	Z	X	Y	Z		
1	0.	2.35000E+01	0.	-5.08776E+07	-1.20648E+08	2.86503E+08	0.	0.	0.		
2	6.08000E+01	0.	1.95800E+01	-1.70880E+09	0.	8.75438E+08	0.	0.	0.		
3	0.	-2.35000E+01	0.	-5.08776E+07	1.20648E+08	2.86503E+08	0.	0.	0.		
4	-9.91000E+00	0.	5.58056E+01	1.81056E+09	0.	-1.44844E+09	0.	0.	0.		
STRUCTURAL ELEMENT	AXIAL FORCE	AXIAL STROKE	AXIAL STIFFNESS	SHEAR FORCE	SHEAR MOMENT	SHEAR FORCE	SHEAR MOMENT	LANDER COORDINATE SYSTEM	LANDER COORDINATE SYSTEM		
1	-3.15006E+08	-5.73075E-03	5.49676E+10	0.	0.	0.	0.	NODE 4	0.		
2	-1.92000E+09	-1.19952E+01	0.	0.	0.	0.	0.	NODE 4	0.		
3	-3.15006E+08	-5.73075E-03	5.49676E+10	0.	0.	0.	0.	NODE 4	0.		
TOTAL ENERGY ABSORBED											
SURFACE COORDINATE SYSTEM			LANDER COORDINATE SYSTEM			SURFACE COORDINATE SYSTEM			LANDER COORDINATE SYSTEM		
X	Y	Z	X	Y	Z	X	Y	Z	X	Y	Z
2.069E+10	0.	-4.940E+09	2.069E+10	0.	-4.940E+09	0.	0.	0.	0.	0.	0.

CPU TIME USAGE TABLE (SEL)

	TIME IN	TIME OUT	TOTAL
INPUT AND INITIALIZATION OVERLAY	50.025	50.030	.01
GEAR INPUT AND ANALYSIS OVERLAY	50.030	50.867	5.80

APPENDIX D

EXAMPLE OF INPUT AND OUTPUT
LANDING LOADS AND MOTIONS PROGRAM

LANDING LEGS AND MOTIONS PROGRAM - LEGGED LANDING
 MASTER AGREEMENT, CONTRACT NAS1-8137, TASK ORDER NUMBER FIV.
 McDONNELL DOUGLAS AERONAUTICS COMPANY - EAST

INPUT DATA

NEW CARDS READ THIS RUN = 1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20
 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 39 40 41 42 43 44 45 46 47 48 49 50 51 52 53 54 55 56 57 58 59 60
 61 62 63 64 65 66 67 68 69 70 71 72 73 74 75 76 77 78 79 80 81 82 83 84 85 86 87 88 89 90 91 92 93 94 95 96 97 98 99 100
 101 102 103 104 105 106 107 108 109 110 111 112 113 114 115 116 117 118 119 120 121 122 123 124 125 126 127 128 129 130 131 132 133 134 135 136 137 138 139 140
 141 142 143 144 145 146 147 148 149 150 151 152 153 154 155 156 157 158 159 160 161 162 163 164 165 166 167 168 169 170 171 172 173 174 175 176 177 178 179 180
 181 182 183 184 185 186 187 188 189 190 191 192 193 194 195 196 197 198 199 200 201 202 203 204 205 206 207 208 209 210 211 212 213 214 215 216 217 218 219 220

CARD
 NC

* * PROGRAM CONTROL DATA * *

1 CASE NC = 306
 2 TIME * 1.000E+01 IFFCT = 10 INLES = 1 IFFRT = 1
 3 ANGLE = 3 NODUT = 0
 4 INDFD = 0 INDFY = 1 INDFZ = 0
 5 INDFAR = 1 INDFYK = 0 INDFZR = 1
 6 HMAX = 1.000E+04 HMIN = 0. HMAX = 0. HMIN = 0. IF = 1
 7 LVAFF = 1 LVAFF = 0 CUTERR = 1.000E+04
 8 WFORC = 1 WFORC = 0 WFORC = 0 WFORC = 0

* * INITIAL CONDITIONS * *

9 ZETA = 0. GRAV = 9.807E+02 GRAVE = 9.837E+02
 10 ANGX = 0. ANGY = 9.900E+00 ANGZ = 0.
 11 WX = 0. WY = 0. WZ = 0.
 12 VELX = -1.146E+03 VEY = 0. VELZ = 0.

* * CENTER BODY DATA * *

13 CEMASS = 1.805E+05
 14 CEIXY = 8.031E+08 CEIYV = 4.592E+08 CEIYZ = 4.966E+08
 15 CEIXY = 0. CEIYZ = 0.

NO SECONDARY OUTPUT POINTS

* * FOOTPAD DATA * *

16 FFMASS = 3.497E+03
 17 FFC(I) = 1.460E+01 1.364E+01 1.805E+01 1.805E+01
 18 FFC(I) = 1.199E+01 1.032E+01 8.558E+00 5.870E+00
 19 ATTPK(I) = 1.270E+06 3.970E+00 5.720E+00
 20 ATTPK(I) = 6.210E+06 5.170E+06 5.170E+06

* * SOIL DATA * *

21 ATYPE = 1
 22 SOIL(I) = 9.100E-01, 5.420E+06, 1.470E+07

* * LEG DATA * *

25 NLEEG = 3 ILEG = J DRAGST = 0.
 200 I = 1 XFP = -5.648E+01 YFP = 0. ZFP = 1.4481E+02
 200 I = 2 XFP = -5.648E+01 YFP = -1.254E+02 ZFP = -7.2391E+01
 200 I = 3 XFP = -5.648E+01 YFP = 1.254E+02 ZFP = -7.2391E+01
 300 I = 1 XPSOB = 2.949E+01 YMSOB = 0. ZMSOB = 1.135E+02
 300 I = 2 XPSOB = 2.949E+01 YMSOB = -9.832E+01 ZMSOB = -5.677E+01
 300 I = 3 XPSOB = 2.949E+01 YMSOB = 9.832E+01 ZMSOB = -5.677E+01
 24 PFCMS = 1.090E+09 1.544E+09 5.000E+09 3.340E+09 3.544E+09
 26 PFCMS = 6.300E+09 1.135E+01 1.415E+01 2.000E+01 2.500E+01
 27 CLTMS = 2.000E+00 5.000E+00 4.000E+00 5.000E+00 6.000E+00
 28 SRGMS = 2.482E+09 2.480E+09 1.023E+09 1.138E+09 1.158E+09
 25 SMTMS = 2.500E+09 2.500E+09 2.500E+09 2.500E+09 2.500E+09
 30 SCMXLS = 2.794E+01 STMXMS = 2.000E+00 SRGUCS = 1.442E+10 SKRTFS = 2.500E+01
 31 IFTMS = 0
 32 PFCMS = 0. COLTMS = 0. GAMS = 1.000E+00 ACTI = 0.
 400 I = 1 XUSCB = -3.139E+01 YUSCB = 2.550E+01 ZUSCB = 5.358E+01
 400 I = 2 XUSCB = -3.139E+01 YUSCB = -2.550E+01 ZUSCB = 9.398E+01
 400 I = 3 XUSCB = -3.139E+01 YUSCB = -9.314E+01 ZUSCB = -2.665E+01
 400 I = 4 XUSCB = -3.139E+01 YUSCB = -8.969E+01 ZUSCB = -6.734E+01
 400 I = 5 XUSCB = -3.139E+01 YUSCB = 9.314E+01 ZUSCB = -2.665E+01
 400 I = 6 XUSCB = -3.139E+01 YUSCB = 8.969E+01 ZUSCB = -6.734E+01
 33 PFCUS = 5.500E+10 1.100E+11 1.650E+11 2.200E+11 2.750E+11
 34 PFTDS = 5.500E+10 1.100E+11 1.650E+11 2.200E+11 2.750E+11
 35 CLCDS = 1.000E+00 2.000E+00 3.000E+00 4.000E+00 5.000E+00
 36 CDTDS = 1.000E+00 2.000E+00 3.000E+00 4.000E+00 5.000E+00
 37 SRGUS = 5.500E+10 5.500E+10 5.500E+10 5.500E+10 5.500E+10
 38 SKRTS = 5.500E+10 5.500E+10 5.500E+10 5.500E+10 5.500E+10
 39 SCMXLS = 5.000E+00 STMXDS = 5.000E+00 SRGUCS = 5.500E+10 SKRTLS = 5.500E+10
 40 IKTDS = 0
 41 FRGUS = 0. COLFDS = 0. GAMS = 1.000E+00

* * MODAL DATA * *

42 MODIN = 3
 200 I = 1 GM = 8.405E+04 OMEGA = 3.459E+01
 300 I = 2 GM = 6.800E+04 OMEGA = 4.617E+01
 500 I = 3 GM = 5.848E+04 OMEGA = 1.010E+02
 600 I = 1 MNX = 8.405E+04 MNY = 0.430E+04 MNZ = 1.971E+04
 600 I = 2 MNX = 2.968E+04 MNY = 5.374E+04 MNZ = 5.417E+04
 600 I = 3 MNX = 5.831E+04 MNY = 3.894E+04 MNZ = 2.171E+04
 700 I = 1 FX = 2.486E+01 PY = -2.423E+05 PZ = 2.463E+05
 700 I = 2 FX = 4.677E+03 PY = 1.108E+05 PZ = 1.003E+05
 700 I = 3 FX = 1.852E+02 PY = -2.333E+06 PZ = 2.321E+06
 800 I = 1 FMSX = 5.360E-02 8.110E-01 1.719E-02
 800 I = 2 FMSX = 2.690E-02 4.005E-01 0.
 800 I = 3 FMSX = 2.690E-02 4.005E-01 0.
 900 I = 1 FMSY = 0.
 900 I = 2 FMSY = 5.182E-01 5.182E-01 3.526E-01
 900 I = 3 FMSY = -2.265E-01 -5.182E-01 -3.526E-01
 1000 I = 1 FMSZ = 1.000E+00 5.621E-01 1.988E-01
 1000 I = 2 FMSZ = 6.077E-01 -3.490E-01 -4.180E-01
 1000 I = 3 FMSZ = 6.077E-01 -3.490E-01 -4.180E-01
 1100 I = 1 FDSX = 0.
 1100 I = 2 FDSX = 0.
 1100 I = 3 FDSX = 0.
 1200 I = 1 FDSY = 0.
 1200 I = 2 FDSY = 0.
 1200 I = 3 FDSY = 7.650E-01 -7.574E-01 9.206E-04

1200	I = 4	FDSY = 0.	0.	7.574E-01	1.850E-02
1200	I = 5	FDSY = -7.650E-01	0.	7.574E-01	-3.206E-01
1200	I = 6	FDSY = 0.	0.	0.	-1.850E-02
1300	I = 1	FDSZ = 8.724E-01	-2.809E-01	1.880E-01	1.880E-01
1300	I = 2	FDSZ = 8.724E-01	-2.809E-01	1.880E-01	1.880E-01
1300	I = 3	FDSZ = 4.297E-01	1.638E-01	5.900E-01	-3.660E-01
1300	I = 4	FDSZ = -1.600E-02	5.900E-01	-6.847E-01	-3.660E-01
1300	I = 5	FDSZ = 4.297E-01	1.638E-01	5.900E-01	-3.660E-01
1300	I = 6	FDSZ = -1.200E-02	5.900E-01	-6.847E-01	-3.660E-01
1700	I = 1	FCGX = 0.	FCGY = 0.	FCGZ = -5.170E-01	
1700	I = 2	FCGX = 3.500E-03	FCGY = 0.	FCGZ = -4.590E-02	
1700	I = 3	FCGX = 0.	FCGY = 0.	FCGZ = 3.489E-01	

LANDER INITIAL CONDITIONS (SURFACE COORDINATE SYSTEM)

INITIAL C.G. POSITION X 79.24 Y 0.00 Z 0.00
 INITIAL C.G. VELOCITY X -1146.00 Y 0.00 Z 0.00
 INITIAL ANGULAR VELOCITY X 0.00 Y 0.00 Z 0.00

TIME = 0. STEP SIZE = 1.000000L-04

CENTER BODY MOTIONS

TRANSLATION - SURFACE COORDINATE SYSTEM ROTATION VELOCITY AND ACCELERATION
 LANDER COORDINATE SYSTEM

	DISP	VEL	ACCEL	PHI	THETA	PSI	U.	V.	W.	X	Y	Z	U.	V.	W.
X	7.924E+01	-1.146E+03	-9.807E+02	0.	9.150E+00	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
Y	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
Z	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.

FOOTPAD MOTIONS - SURFACE COORDINATE SYSTEM

FOOTPAD	DISP	VEL	ACCEL	DISP	VEL	ACCEL	DISP	VEL	ACCEL
1	4.743E+01	-1.146E+03	-9.807E+02	0.	0.	0.	1.521E+02	0.	0.
2	1.159E+01	-1.146E+03	-9.807E+02	-1.254E+02	0.	0.	6.240E+01	0.	0.
3	1.159E+01	-1.146E+03	-9.807E+02	1.254E+02	0.	0.	6.240E+01	0.	0.

ACCELERATIONS IN EARTH G'S - LANDER COORDINATE SYSTEM

CENTER OF GRAVITY X Y Z
 -9.863E-01 0. -1.650E-01

LANDING GEAR STRUT INFORMATION

LEG	CRUSH	MAIN STRUT	LOAD	STROKE	LOAD	STROKE	LOAD	STROKE
1	0.	0.	0.	0.	0.	0.	0.	0.
2	0.	0.	0.	0.	0.	0.	0.	0.
3	0.	0.	0.	0.	0.	0.	0.	0.

TIME = 1.000000E-03 STEP SIZE = 1.000000L-04

CENTER BODY MOTIONS

TRANSLATION - SURFACE COORDINATE SYSTEM ROTATION VELOCITY AND ACCELERATION
 LANDER COORDINATE SYSTEM

	DISP	VEL	ACCEL	PHI	THETA	PSI	U.	V.	W.	X	Y	Z	U.	V.	W.
X	7.810E+01	-1.142E+03	1.166E+04	0.	9.150E+00	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
Y	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
Z	-1.730E-04	-7.048E-01	-1.779E+03	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.

FOOTPAD MOTIONS - SURFACE COORDINATE SYSTEM

FOOTPAD	DISP	VEL	ACCEL	DISP	VEL	ACCEL	DISP	VEL	ACCEL
1	4.743E+01	-1.142E+03	1.166E+04	0.	0.	0.	0.	0.	0.
2	1.159E+01	-1.142E+03	1.166E+04	-1.254E+02	0.	0.	6.240E+01	0.	0.
3	1.159E+01	-1.142E+03	1.166E+04	1.254E+02	0.	0.	6.240E+01	0.	0.

FOOTPAD 1 4.629E+01 -1.149E+03 -1.100E+04 0. 1.521E+02 0. JORSTLU 3.006E+04
 FOOTPAD 2 1.052E+01 -1.023E+03 -1.047E+04 -1.254E+02 -0.722E+01 -1.934E+05 -6.208E+01 -3.045E+01 -1.000E+02
 FOOTPAD 3 1.052E+01 -1.023E+03 -1.047E+04 1.254E+02 0.722E+01 1.934E+05 -0.208E+01 -3.045E+01 -1.000E+02

ACCELERATIONS IN EARTH G'S - LANDER COORDINATE SYSTEM

CENTER OF GRAVITY 1.000E+01 0. 1.739E+01

LANDING GEAR STRUT INFORMATION

LEG	CRUSH	LOAD	STROKE	LOAD	STROKE	LOAD	STROKE
1	0.	MAIN STRUT -2.072E+03	-0.844E-03	LRAG STRUT 1 -6.551E+01	-1.818E-03	URAG STRUT 2 -6.551E+07	-1.818E-03
2	6.632E-01	MAIN STRUT -1.631E+08	-6.571E-02	LRAG STRUT 3 -1.351E+03	-2.451E-02	DRAG STRUT 4 -1.022E+09	-1.674E-02
3	6.632E-01	MAIN STRUT -1.631E+08	-6.571E-02	LRAG STRUT 5 -1.351E+03	-2.451E-02	DRAG STRUT 6 -1.022E+09	-1.674E-02

TIME = 2.000000E-03 STEP SIZE = 1.000000E-04

CENTER BODY MOTIONS

TRANSLATION - SURFACE COORDINATE SYSTEM ROTATION

	DISP	VEL	ACCEL	DISP	VEL	ACCEL	DISP	VEL	ACCEL
X	7.696E+01	-1.122E+03	3.478E+04	0.	0.	0.	1.521E+02	0. JORSTLU	3.006E+04
Y	0.	0.	0.	0.	0.	0.	0.	0.	0.
Z	-1.091E-03	-3.092E+03	-5.476E+03	0.	0.	0.	0.	0.	0.

FOOTPAD MOTIONS - SURFACE COORDINATE SYSTEM

	DISP	VEL	ACCEL	DISP	VEL	ACCEL	DISP	VEL	ACCEL
FOOTPAD 1	4.513E+01	-1.158E+03	-1.504E+04	0.	0.	0.	1.521E+02	0. JORSTLU	3.006E+04
FOOTPAD 2	9.000E+00	-8.020E+02	-9.775E+04	-1.254E+02	-1.131E+02	-5.305E+05	-6.212E+01	-6.122E+01	-2.892E+02
FOOTPAD 3	9.000E+00	-8.020E+02	-9.775E+04	1.254E+02	1.131E+02	5.305E+05	-6.212E+01	-6.122E+01	-2.892E+02

ACCELERATIONS IN EARTH G'S - LANDER COORDINATE SYSTEM

CENTER OF GRAVITY 3.582E+01 0. 3.275E-01

LANDING GEAR STRUT INFORMATION

LEG	CRUSH	LOAD	STROKE	LOAD	STROKE	LOAD	STROKE
1	0.	MAIN STRUT 9.600E+06	3.840E-03	LRAG STRUT 1 -1.118E+09	-2.035E-03	URAG STRUT 2 -1.118E+08	-2.035E-03
2	1.030E+00	MAIN STRUT -6.178E+08	-2.448E-01	LRAG STRUT 3 -3.554E+09	-6.443E-02	DRAG STRUT 4 -2.499E+09	-5.391E-02
3	1.030E+00	MAIN STRUT -6.178E+08	-2.448E-01	LRAG STRUT 5 -3.554E+09	-6.443E-02	DRAG STRUT 6 -2.499E+09	-5.391E-02

TIME = 3.000000E-03 STEP SIZE = 1.000000E-04

CENTER BODY MOTIONS

TRANSLATION - SURFACE COORDINATE SYSTEM ROTATION

	DISP	VEL	ACCEL	DISP	VEL	ACCEL	DISP	VEL	ACCEL
X	7.458E+01	-1.100E+03	3.223E+04	0.	0.	0.	1.521E+02	0. JORSTLU	3.006E+04
Y	0.	0.	0.	0.	0.	0.	0.	0.	0.
Z	-7.924E-03	-8.066E+03	-6.457E+03	0.	0.	0.	0.	0.	0.

TIME = 9.900000E-02 STEP SIZE = 1.000000E-04

CENTER BODY MOTIONS

TRANSLATION - SURFACE COORDINATE SYSTEM ROTATION

VELOCITY AND ACCELERATION LANDER COORDINATE SYSTEM			EULER ANGLES		
X	Y	Z	PHI	THETA	PSI
4.730E+01	3.743E+01	-1.125E+03	0.	-1.847E+00	0.
0.	0.	0.	0.	0.	0.
1.015E+01	-2.869E+02	6.009E+03	0.	-1.031E+01	-3.822E+02

FOOTPAD MOTIONS - SURFACE COORDINATE SYSTEM

SURFACE COORDINATE SYSTEM			VELOCITY AND ACCELERATION		
FOOTPAD	DISP	VEL	ACCEL	DISP	VEL
1	8.155E+00	2.912E-04	-5.823E+00	0.	0.
2	6.339E+00	6.797E+00	-1.497E+03	-1.297E+02	1.031E+00
3	8.398E+00	6.757E+00	-1.456E+03	1.297E+02	-1.031E+00

ACCELERATIONS IN EARTH G'S - LANDER COORDINATE SYSTEM

LANDER COORDINATE SYSTEM		
X	Y	Z
-9.464E-01	0.	6.237E+00

LANDING GEAR STRUT INFORMATION

LEG	CRUSH	LOAD	STROKE	LOAD	STROKE	LOAD	STROKE
1	2.634E+00	MAIN STRUT	0.	-1.170E+01	DRAG STRUT 1	-2.007E+00	-3.645E-05
2	2.339E+00	MAIN STRUT	0.	-1.193E+01	DRAG STRUT 3	1.604E+08	2.917E-03
3	2.339E+00	MAIN STRUT	0.	-1.193E+01	DRAG STRUT 5	1.604E+08	2.917E-03

TIME = 1.000000E-01 STEP SIZE = 1.000000E-04

CENTER BODY MOTIONS

TRANSLATION - SURFACE COORDINATE SYSTEM ROTATION

VELOCITY AND ACCELERATION LANDER COORDINATE SYSTEM			EULER ANGLES		
X	Y	Z	PHI	THETA	PSI
4.730E+01	3.743E+01	-1.125E+03	0.	-1.847E+00	0.
0.	0.	0.	0.	0.	0.
9.948E+00	-1.942E+02	1.902E+04	0.	-1.031E+01	-3.822E+02

FOOTPAD MOTIONS - SURFACE COORDINATE SYSTEM

SURFACE COORDINATE SYSTEM			VELOCITY AND ACCELERATION		
FOOTPAD	DISP	VEL	ACCEL	DISP	VEL
1	8.155E+00	9.707E-01	-2.775E+03	0.	0.
2	8.403E+00	2.367E+00	-1.145E+03	-1.297E+02	1.031E+00
3	8.403E+00	2.367E+00	-1.145E+03	1.297E+02	-1.031E+00

ACCELERATIONS IN EARTH G'S - LANDER COORDINATE SYSTEM

LANDER COORDINATE SYSTEM		
X	Y	Z
-9.299E-01	0.	1.654E+01

LANDING GEAR STRUT INFORMATION

LEG	CRUSH	LOAD	STROKE	LOAD	STROKE	LOAD	STROKE
1	2.634E+00	MAIN STRUT	0.	-1.170E+01	DRAG STRUT 1	-4.065E+07	-7.359E-04
2	2.339E+00	MAIN STRUT	0.	-1.188E+01	DRAG STRUT 3	1.639E+08	2.934E-03
3	2.339E+00	MAIN STRUT	0.	-1.188E+01	DRAG STRUT 5	1.639E+08	2.934E-03

 TIME = 1.010000E-01 STEP SIZE = 1.000000E-04

CENTER BODY MOTIONS

TRANSLATION - SURFACE COORDINATE SYSTEM ROTATION VELOCITY AND ACCELERATION
 LANDER COORDINATE SYSTEM

	DISP	VEL	ACCEL	PHI	U.	VEL	ACCEL	DISP	VEL	ACCEL
X	4.737E+01	2.443E+01	-1.904E+03	0.	0.	1.600E+02	1.180E+02	1.600E+02	1.180E+02	3.030E+04
Y	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
Z	9.705E+00	-1.698E+02	2.802E+04	THETA	-1.670E+00	Y	-1.000E+01	2.307E+03	Z	0.
				PSI	0.	Z	0.	0.		

FOOTPAD MOTIONS - SURFACE COORDINATE SYSTEM

FOOTPAD	DISP	VEL	ACCEL	DISP	VEL	ACCEL	DISP	VEL	ACCEL
1	8.157E+00	9.852E-01	-3.527E+03	0.	0.	0.	1.600E+02	1.180E+02	3.030E+04
2	8.440E+00	2.509E+00	2.637E+03	-1.256E+02	-1.447E+01	5.068E+04	-6.537E+01	1.597E+02	-7.723E+04
3	8.440E+00	2.509E+00	2.637E+03	1.256E+02	1.447E+01	-5.068E+04	-6.537E+01	1.597E+02	-7.723E+04

ACCELERATIONS IN EARTH G'S - LANDER COORDINATE SYSTEM

CENTER OF GRAVITY X Y Z
 -9.802E-01 0. 2.924E+01

LANDING GEAR STRUT INFORMATION

LEG	CRUSH	LOAD	STROKE	LOAD	STROKE	LOAD	STROKE
1	2.834E+00	MAIN STRUT	0.	-3.779E+01	DRAG STRUT 1	-5.790E+07	3.094E-03
2	2.339E+00	MAIN STRUT	0.	-1.188E+01	DRAG STRUT 3	-4.091E+06	-7.439E-03
3	2.339E+00	MAIN STRUT	0.	-1.188E+01	DRAG STRUT 5	-4.091E+06	-7.439E-03
					DRAG STRUT 2	-5.790E+07	3.094E-03
					DRAG STRUT 4	4.327E+08	7.860E-03
					DRAG STRUT 6	4.327E+08	7.860E-03

SUMMARY INFORMATION

MAXIMUM MAIN STRUT STROKES

STRUT	TIME	STROKE	TENSION	TIME	STROKE
1	5.720E-02	-1.899E+01	1.350E-02	7.136E-02	0.
2	3.520E-02	-1.368E+01	0.	0.	0.
3	3.520E-02	-1.368E+01	0.	0.	0.

MAXIMUM DRAG STRUT STROKES

STRUT	TIME	STROKE	TENSION	TIME	STROKE
1	3.070E-02	-6.536E-02	3.260E-02	4.593E-02	0.
2	3.070E-02	-6.536E-02	3.260E-02	4.593E-02	0.
3	2.000E-03	-6.439E-02	5.990E-02	3.005E-02	0.
4	4.400E-03	-6.078E-02	3.710E-02	3.855E-02	0.
5	2.000E-03	-6.439E-02	5.990E-02	3.005E-02	0.
6	4.400E-03	-6.078E-02	3.710E-02	3.855E-02	0.

END OF CASE - TIME LIMIT

CASE NO 366 RAN FOR 58.524 CF SECONDS

APPENDIX E
OPERATING INSTRUCTIONS
CENTER BODY LANDING LOADS PROGRAM

E.1 Introduction - The Center Body Landing Loads Program retrieves and lists, for specific points in time, the information placed on magnetic tape by the Landing Loads and Motions Program. Information continuously stored on tape consists of landing gear strut forces acting on the center body; angular positions, velocities, and accelerations and the translational accelerations of the center body center of gravity; and the displacements, velocities, and accelerations of the normal modes employed in representing the flexible center body structure. The Center Body Landing Loads Program uses the information on this tape, in addition to other required input data, to obtain the distribution of landing gear strut loads and inertia loads throughout the center body structure for the selected points in time. This load information may then be used in conjunction with the Center Body Option of the Structural Analysis Program to determine the internal member load distribution for the time point of interest.

E.2 Input Data - A number of input data cards are required by the Center Body Landing Loads Program. These properly initialize the program, define the mass properties of the center body, and define the points in time during the landing time history at which the loads are to be determined. The required format for these cards is shown in Figure E-1 while the input variables are defined in Figure E-2. Also indicated in Figure E-2 is whether the various input quantities must be input as integer or floating point numbers. All integer numbers and floating point numbers input with an E format must be right justified. Floating point numbers input with an F format may appear anywhere inside their allotted field on the data card.

In addition to the required card input, two magnetic tapes are required. The mode shape data and joint coordinate information is required on TAPE4. This tape is generated when performing the modal analysis on the center body structure with the Structural Analysis Program. Secondly, the time history data obtained from the Landing Loads and Motions Program must be available on TAPE3. Since the time history data on this tape corresponds to points in time when normal printed output was obtained from the Landing Loads and Motions Program, only center body loads at these points in time may be obtained with the Center Body Landing Loads Program.

COLUMN:	5	6	10	11	15	16	20	21	25	26	30	31	35	36	40	41	45	46	50	
1																				
	NOJTS	NOLEG	NOMODE	NOTIME	NOSETS															
	MODE(1)	MODE(2)	MODE(3)	MODE(4)	MODE(5)															
	JOTMS(1)	JOTMS(2)	JOTMS(3)	JOTMS(4)	JOTMS(5)															
	JOTDS(1)	JOTDS(2)	JOTDS(3)	JOTDS(4)	JOTDS(5)	JOTDS(6)	JOTDS(7)	JOTDS(8)	JOTDS(9)	JOTDS(10)										
	ZETA		GRAV																	
	TYM(I)		TYM(I+1)		TYM(I+2)		TYM(I+3)		TYM(I+4)											
	AMASS(I)		AMASS(I+1)		AMASS(I+2)		AMASS(I+3)		AMASS(I+4)											

FIGURE E-1 INPUT DATA FORMAT - CENTER BODY LANDING LOADS PROGRAM

INPUT VARIABLE	TYPE NUMBER*	VARIABLE DEFINITION
NOJTS	I	NUMBER OF JOINTS IN CENTER BODY STRUCTURAL IDEALIZATION (74 MAXIMUM)
NOLEG	I	NUMBER OF LEGS (5 MAXIMUM)
NOMODE	I	NUMBER OF MODES INCLUDED IN ANALYSIS (5 MAXIMUM)
NOTIME	I	NUMBER OF TIME POINTS TO BE READ FROM SECONDARY TIME HISTORY TAPE (TAPE3) OBTAINED FROM LANDING LOADS AND MOTIONS PROGRAM (20 MAXIMUM)
NOSETS	I	NUMBER OF NORMAL MODE SETS APPEARING ON TAPE (TAPE4) OBTAINED DURING NORMAL MODE ANALYSIS WITH THE STRUCTURAL ANALYSIS PROGRAM
MODE(I)	I	MODE NUMBER OF I TH NORMAL MODE OBTAINED FROM THE STRUCTURAL ANALYSIS PROGRAM (5 MAXIMUM)
JOTMS(I)	I	JOINT NUMBER LOCATION OF I TH MAIN STRUT (5 MAXIMUM)
JOTDS(I)	I	JOINT NUMBER LOCATION OF I TH DRAG STRUT (10 MAXIMUM)
ZETA	F	GROUND SLOPE
GRAV	F	LOCAL ACCELERATION OF GRAVITY
TYM(I)	F	TIMES, IN ASCENDING ORDER, AT WHICH CENTER BODY LOADS ARE TO BE DETERMINED (20 MAXIMUM, 5 TO A DATA CARD)
AMASS(I)	F	MASS ASSOCIATED WITH I TH STRUCTURAL JOINT. (74 MAXIMUM, 5 TO A DATA CARD)

*I - INTEGER NUMBER; F - FLOATING POINT NUMBER

FIGURE E-2 INPUT DATA - CENTER BODY LANDING LOADS PROGRAM

The use of the input indicators NOSETS and MODE(I) require some additional comments. The modal analysis output obtained from the Structural Analysis Program is placed on magnetic tape in data sets each containing information for five modes. The first set contains the lowest five elastic modes, the second set contains the next five modes, etc. The input indicator NOSETS defines the number of these modal data sets that appear on TAPE4. The modes in the first data set are numbered 1 through 5, those in the second data set 6 through 10, etc. MODE(I) defines the mode number of the Ith mode which is required by the Center Body Landing Loads Program.

E.3 Output Data - For a selected point in time the landing loads at each of the joints used in the center body structural idealization are printed. These loads are the combination of the inertia loads, gravity loads, and landing gear strut loads. The components of the landing loads are given in the Lander Coordinate System and thus may be directly input to the Center Body Option of the Structural Analysis Program. In addition, the sum of forces and moments about the center of gravity are given for information purposes.

E.4 Program Operation - An example of the output from the Center Body Landing Loads Program is given on the following pages. This information was required during the analysis of the Task Order Three lander, Section 6.2.3, at a time of 0.011 seconds. This time was selected for determining internal loads distribution in the lander center body using the Center Body Option of the Structural Analysis Program as discussed in Section 6.2.4.

A listing of the Center Body Landing Loads Program follows the example output.

GENIER BUDY LAINING...LOALS PROGRAM
 MASTER AGREEMENT, CONTRAKT NASI-8107, TASK UKUR FIVE
 McDONNELL DUGELAS AERONAUTICALS COMPANY - LASI

NOJTS = 33
 NCLG = 3
 NCMODE = 3
 NOILINE = 4
 NCSETS = 4
 MODE = 1 3 8
 JCTVS = 5 15 25
 JOTLS = 2 7 12 17 27 42
 ZETA = 0.
 GRAY = 9.807E+02
 TYM = 1.100E-02
 MASS = 3.789E+03 6.039E+03 5.549E+03 2.490E+03 3.483E+03 3.789E+03 3.533E+03 1.079E+03 1.736E+03
 3.789E+03 6.039E+03 5.549E+03 2.127E+03 3.483E+03 3.789E+03 3.533E+03 2.046E+03 1.736E+03
 3.789E+03 6.039E+03 5.549E+03 2.127E+03 3.483E+03 3.789E+03 3.533E+03 1.675E+03 1.736E+03
 3.487E+04 3.270E+04 3.578E+03

JOINT COORDINATES

JOINT	X	Y	Z
1	1.359E+01	3.172E+01	6.578E+01
2	-2.807E+01	3.172E+01	6.578E+01
3	1.359E+01	0.	6.578E+01
4	-2.807E+01	0.	6.578E+01
5	2.949E+01	0.	1.335E+02
6	1.359E+01	-3.172E+01	6.578E+01
7	-2.807E+01	-3.172E+01	6.578E+01
8	1.359E+01	-6.093E+01	3.518E+01
9	-1.731E+01	-6.093E+01	3.518E+01
10	-2.807E+01	-6.093E+01	3.518E+01
11	1.359E+01	-9.014E+01	-1.542E+01
12	-2.807E+01	-9.014E+01	-1.542E+01
13	1.359E+01	-7.430E+01	-4.288E+01
14	-2.807E+01	-7.430E+01	-4.288E+01
15	2.949E+01	-4.832E+01	-5.677E+01
16	1.359E+01	-5.842E+01	-7.036E+01
17	-2.807E+01	-5.842E+01	-7.036E+01
18	1.359E+01	0.	-7.036E+01
19	-1.731E+01	0.	-7.036E+01
20	-2.807E+01	0.	-7.036E+01
21	1.359E+01	5.842E+01	-7.036E+01
22	-2.807E+01	5.842E+01	-7.036E+01
23	1.359E+01	7.430E+01	-4.288E+01
24	-2.807E+01	7.430E+01	-4.288E+01
25	2.949E+01	9.832E+01	-5.677E+01
26	1.359E+01	9.014E+01	-1.542E+01
27	-2.807E+01	9.014E+01	-1.542E+01
28	1.359E+01	6.093E+01	3.518E+01
29	-1.731E+01	6.093E+01	3.518E+01
30	-2.807E+01	6.093E+01	3.518E+01
31	1.359E+01	0.	0.
32	-1.731E+01	0.	0.
33	0.	0.	0.

MODE SHAPES

MODE = 1

JCINT	X	Y	Z
1	-7.546E-04	-3.097E-04	8.734E-01
2	-7.585E-04	-4.304E-04	8.744E-01
3	-2.552E-04	4.316E-07	5.676E-01
4	-7.777E-04	5.366E-07	8.973E-01
5	-5.363E-02	-2.449E-05	1.000E+00
6	-7.646E-04	3.105E-04	8.733E-01
7	-7.586E-04	4.315E-04	8.723E-01
8	-1.597E-03	6.427E-01	5.000E-01
9	-1.561E-03	6.428E-01	4.936E-01
10	-1.578E-03	6.401E-01	5.016E-01
11	-2.957E-03	7.601E-01	4.329E-01
12	-2.944E-03	7.651E-01	4.297E-01
13	1.105E-04	4.217E-01	6.375E-01
14	3.700E-04	3.938E-01	2.152E-01
15	2.690E-02	2.265E-01	8.072E-01
16	3.688E-03	1.453E-03	-5.483E-03
17	3.670E-03	9.660E-04	-1.197E-02
18	3.149E-03	-4.452E-05	-6.133E-01
19	3.075E-03	-4.447E-05	-6.139E-01
20	3.110E-03	-4.448E-05	-6.070E-01
21	3.688E-03	-1.542E-03	-5.436E-03
22	3.670E-03	-1.055E-03	-1.192E-02
23	1.105E-04	-4.218E-01	6.376E-01
24	3.701E-04	-3.939E-01	2.153E-01
25	2.691E-02	-2.266E-01	8.077E-01
26	-2.956E-03	-7.601E-01	4.330E-01
27	-2.944E-03	-7.651E-01	4.297E-01
28	-1.597E-03	-6.427E-01	5.001E-01
29	-1.561E-03	-6.428E-01	4.936E-01
30	-1.578E-03	-6.400E-01	5.017E-01
31	-2.033E-05	3.199E-05	-6.164E-01
32	-2.024E-05	3.201E-05	-6.178E-01
33	-2.029E-05	3.200E-05	-6.170E-01

MODE = 3.

JCINT	X	Y	Z
1	-3.986E-01	9.666E-03	2.972E-01
2	-3.991E-01	9.616E-03	-2.952E-01
3	-3.863E-01	8.568E-03	3.151E-01
4	-3.873E-01	8.993E-03	-2.925E-01
5	-8.116E-01	9.796E-03	5.621E-01
6	-3.706E-01	7.421E-03	2.827E-01
7	-3.711E-01	8.352E-03	-2.801E-01
8	3.227E-01	2.014E-01	1.648E-01
9	3.229E-01	-1.201E-01	-7.311E-02
10	3.227E-01	-2.356E-01	-1.437E-01
11	9.968E-01	7.294E-01	-1.466E-01
12	1.000E+00	-7.574E-01	1.636E-01
13	1.934E-01	3.797E-01	-3.497E-01
14	1.939E-01	-3.838E-01	3.797E-01
15	4.005E-01	5.182E-01	-3.490E-01
16	-6.157E-01	6.476E-03	5.631E-01
17	-6.154E-01	-5.334E-03	5.972E-01
18	-6.196E-01	-4.204E-03	-1.795E-01
19	-6.201E-01	2.185E-03	1.317E-01
20	-6.156E-01	4.256E-03	2.581E-01
21	-5.918E-01	-1.487E-02	-5.484E-01
22	-5.916E-01	1.363E-02	5.820E-01
23	2.035E-01	-3.824E-01	-3.384E-01
24	2.041E-01	3.861E-01	3.631E-01
25	4.217E-01	-5.256E-01	-3.335E-01
26	9.961E-01	-7.297E-01	-1.394E-01

27 9.963E-01 7.525E-01 1.559E-01
 28 3.065E-01 -1.995E-01 1.708E-01
 29 3.067E-01 1.105E-01 -7.064E-02
 30 3.065E-01 2.321E-01 -1.506E-01
 31 3.487E-03 2.525E-03 -1.674E-01
 32 3.487E-03 -1.777E-03 1.185E-01
 33 3.487E-03 6.459E-04 -4.334E-02

MODE = A

JCANT	X	Y	Z
1	3.304E-03	3.106E-03	2.097E-01
2	3.194E-03	1.925E-03	1.880E-01
3	3.960E-03	1.769E-05	1.829E-01
4	3.053E-03	1.456E-05	1.688E-01
5	-1.710E-02	1.106E-05	1.982E-01
6	3.308E-03	-3.135E-05	2.097E-01
7	3.198E-03	-1.896E-03	1.880E-01
8	2.953E-02	-4.351E-02	2.411E-01
9	2.920E-02	5.996E-02	2.168E-01
10	2.950E-02	-3.485E-02	1.972E-01
11	5.572E-02	1.001E+00	-3.972E-01
12	5.538E-02	5.204E-01	-3.665E-01
13	-1.682E-03	5.023E-01	-6.602E-01
14	-1.222E-03	4.614E-01	-6.305E-01
15	9.217E-03	3.564E-01	-4.180E-01
16	-5.249E-02	2.439E-02	-9.571E-01
17	-5.804E-02	1.849E-02	-8.847E-01
18	-5.859E-02	-3.830E-06	2.989E-01
19	-5.793E-02	-3.325E-06	3.214E-01
20	-5.813E-02	-3.144E-06	2.984E-01
21	-5.848E-02	-2.406E-02	-5.571E-01
22	-2.804E-02	-1.820E-02	-8.847E-01
23	-1.680E-03	-5.023E-01	-6.862E-01
24	-1.221E-03	-4.614E-01	-6.305E-01
25	9.214E-03	-3.563E-01	-4.180E-01
26	5.572E-02	-1.001E+00	-3.972E-01
27	5.538E-02	-9.204E-01	-3.660E-01
28	2.953E-02	4.354E-04	2.411E-01
29	2.920E-02	5.997E-02	2.168E-01
30	2.950E-02	-3.484E-02	1.972E-01
31	2.840E-04	-5.769E-06	3.369E-01
32	2.838E-04	-6.462E-06	3.695E-01
33	2.840E-04	-6.002E-06	3.485E-01

TIME HISTORY DATA

TIME = 1.180E+02
 XSDI = 1.009E+04 YSDI = 0. ZSDI = 6.10E+03
 PHI = 0. THIA = 1.539E-01 FSI = L.
 WX = 0. WY = -2.179E+00 WZ = 0.
 WXC = 0. WYC = -1.941E+02 WZC = 0.
 MCDE = 1. GC = 1.117E+00 GCU = 0.00E+00
 MCDE = 2. GC = 2.522E-01 GCU = 1.720E+00
 MCDE = 3. GC = -2.004E-02 GCU = 1.000E+01 GULL = 1.850E+04

MAIN STRUT LOADS

I = 1 FMSX = -1.679E+07 FMSY = 0. FMSZ = 0.00E+00
 I = 2 FMSX = 5.986E+08 FMSY = 3.883E+02 FMSZ = 2.192E+00
 I = 3 FMSX = 9.546E+08 FMSY = -3.803E+00 FMSZ = 2.052E+00

DRAG STRUT LOADS

I = 1 FDSX = -1.371E+08 FDSY = -1.206E+08 FDSZ = 2.7E+00
 I = 2 FDSX = -1.371E+08 FDSY = 1.206E+08 FDSZ = 2.7E+00
 I = 3 FDSX = 1.694E+08 FDSY = 3.372E+08 FDSZ = 4.059E+00
 I = 4 FDSX = -1.480E+08 FDSY = -4.944E+08 FDSZ = -4.844E+00
 I = 5 FDSX = 1.694E+08 FDSY = -3.372E+08 FDSZ = 4.059E+00
 I = 6 FDSX = -1.480E+08 FDSY = 4.944E+08 FDSZ = -4.844E+00

CENTER BODY LANDING LOADS DISTRIBUTION

JOINT	X	Y	Z
1	1.801E+07	-7.227E+03	4.612E+07
2	-1.096E+06	-1.273E+08	3.736E+08
3	1.737E+07	2.084E+05	5.259E+07
4	1.139E+07	-1.549E+05	4.403E+07
5	4.310E+06	2.316E+05	5.892E+07
6	1.673E+07	4.511E+05	4.572E+07
7	-1.086E+08	1.265E+08	3.742E+08
8	-2.922E+06	7.703E+07	-2.389E+06
9	-1.675E+06	3.738E+07	7.285E+06
10	-1.638E+06	3.221E+07	1.003E+07
11	-2.505E+07	3.671E+07	2.372E+07
12	1.282E+06	3.436E+08	5.363E+08
13	-5.850E+07	2.191E+07	1.500E+07
14	-3.531E+07	1.748E+06	3.324E+07
15	9.359E+08	3.971E+08	2.359E+08
16	-1.000E+08	-1.351E+06	1.254E+06
17	-3.094E+06	-4.965E+08	4.452E+07
18	-1.017E+06	-1.153E+05	-1.479E+08
19	-6.288E+07	3.229E+04	-7.277E+07
20	-4.657E+07	4.866E+04	-4.652E+07
21	-9.541E+07	1.125E+06	1.633E+06
22	-3.084E+08	4.968E+08	4.387E+07
23	-5.825E+07	-2.139E+07	1.528E+07
24	-3.516E+07	-1.716E+06	3.307E+07
25	9.404E+06	-3.973E+08	2.352E+08
26	-2.514E+07	-3.606E+07	2.393E+07
27	-1.280E+06	-3.448E+08	5.366E+08
28	-3.313E+06	-7.698E+07	-2.225E+06
29	-2.086E+06	-3.74E+07	7.221E+06
30	-2.033E+06	-3.224E+07	9.952E+06
31	-3.454E+06	6.454E+05	-1.370E+09
32	-3.276E+08	-3.646E+05	-1.040E+09
33	-3.562E+07	2.041E+04	-1.290E+08

LANDING LOADS SUMMED AT C.G.

FX = -2.277E+04
FY = -9.633E+03
FZ = -2.316E+04
TX = 9.188E+05
TY = -2.557E+09
TZ = 2.758E+06

C	PROGRAM CBLL(INPUT,OUTPUT,TAPE5=INPUT,TAPE6=OUTPUT,TAPE3,TAPE4)	CBLL 10
C		CBLL 20
C	CENTER BODY LANDING LOADS PROGRAM	CBLL 30
C	MASTER AGREEMENT, CONTRACT NAS1-8137, TASK ORDER NUMBER FIVE	CBLL 40
C	MCDONNELL DOUGLAS ASTRONAUTICS COMPANY - EAST	CBLL 50
C		CBLL 60
C	THIS PROGRAM COMBINES THE EFFECTS OF THE LANDING GEAR STRUT LOADS	CBLL 70
C	AND INERTIA LOADS AND OUTPUTS THESE FOR THE STRUCTURAL ANALYSIS	CBLL 80
C	PROGRAM	CBLL 90
C		CBLL 100
C	THE FOLLOWING TAPE ARE RFOQUIRED -	CBLL 110
C	1. TAPE3 - TIME HISTORY INFORMATION FROM THE LANDING LOADS	CBLL 120
C	AND MOTIONS PROGRAM	CBLL 130
C	2. TAPE4 - MODAL DATA FROM THE STRUCTURAL ANALYSIS PROGRAM	CBLL 140
C		CBLL 150
C	DIMENSION DUMMY(75),XJT(80),YJT(80),ZJT(80),PHIX(80,30),	CBLL 160
*	PHIY(80,30),PHIZ(80,30),AMASS(80),XPHI(80,5),	CBLL 170
*	YPHI(80,5),ZPHI(80,5),MODE(5),JOTMS(5),JOTDS(10),	CBLL 180
*	TYM(20),GC(5),GCD(5),GCDD(5),FMSX(5),FMSY(5),FMSZ(5),	CBLL 190
*	FDSX(10),FDSY(10),FDSZ(10),DC(3,3),FX(80),FY(80),FZ(80),	CBLL 200
*	FXJT(80),FYJT(80),FZJT(80),	CBLL 210
*	SAX(80),SAY(80),SAZ(80)	CBLL 220
C		CBLL 230
C	PROGRAM INITIALIZATION DATA	CBLL 240
C		CBLL 250
C	READ(5,500)NOJTS,NOLEG,NOMODE,NOTIME,NOSETS	CBLL 260
C	READ(5,500)(MODE(I),I=1,NOMODE)	CBLL 270
C	READ(5,500)(JOTMS(I),I=1,NOLEG)	CBLL 280
C	NODS=2*NOLEG	CBLL 290
C	READ(5,500)(JOTDS(I),I=1,NODS)	CBLL 300
C	READ(5,501)ZETA,GRAV	CBLL 310
C	READ(5,501)(TYM(I),I=1,NOTIME)	CBLL 320
C	READ(5,501)(AMASS(I),I=1,NOJTS)	CBLL 330
C		CBLL 340
C	PRINT PROGRAM INITIALIZATION DATA	CBLL 350
C		CBLL 360
C	WRITE(6,600)	CBLL 370
C	WRITE(6,601)NOJTS,NOLEG,NOMODE,NOTIME,NOSETS	CBLL 380
C	WRITE(6,602)(MODE(I),I=1,NOMODE)	CBLL 390
C	WRITE(6,603)(JOTMS(I),I=1,NOLEG)	CBLL 400
C	WRITE(6,604)(JOTDS(I),I=1,NODS)	CBLL 410
C	WRITE(6,605)ZETA,GRAV	CBLL 420
C	WRITE(6,607)(TYM(I),I=1,NOTIME)	CBLL 430
C	WRITE(6,606)(AMASS(I),I=1,NOJTS)	CBLL 440
C		CBLL 450
C	READ MODAL DATA FROM TAPE4	CBLL 460
C		CBLL 470
C	REWIND 4	CBLL 480
C	I1=0	CBLL 490
C	I2=0	CBLL 500
C	I3=0	CBLL 510
C	I4=0	CBLL 520
C	I5=0	CBLL 530
150	CONTINUE	CBLL 540
	READ(4,400)MMM,(DUMMY(I),I=1,5)	CBLL 550

155	CONTINUE	CBLL 560
	IF(MMM.EQ.21)GO TO 150	CBLL 570
	IF(MMM.EQ.121)GO TO 150	CBLL 580
	IF(MMM.EQ.221)GO TO 160	CBLL 590
	IF(MMM.EQ.321)GO TO 165	CBLL 600
	IF(MMM.EQ.421)GO TO 156	CBLL 610
	IF(MMM.EQ.521)GO TO 150	CBLL 620
	IF(MMM.EQ.721)GO TO 150	CBLL 630
	WRITE(6,404)	CBLL 640
	STOP	CBLL 650
156	CONTINUE	CBLL 660
	I3=0	CBLL 670
	I4=I4+5	CBLL 680
	I5=I5+1	CBLL 690
	IF(I5.EQ.NOSETS)GO TO 200	CBLL 700
	CALL SKFILE(4,1)	CBLL 710
	GO TO 150	CBLL 720
C		CBLL 730
C	JOINT COORDINATES	CBLL 740
C		CBLL 750
160	CONTINUE	CBLL 760
	IF(I5.NE.0)GO TO 150	CBLL 770
	I1=I1+1	CBLL 780
	XJT(I1)=DUMMY(3)	CBLL 790
	YJT(I1)=DUMMY(1)	CBLL 800
	ZJT(I1)=DUMMY(2)	CBLL 810
	GO TO 150	CBLL 820
C		CBLL 830
C	MODE SHAPES	CBLL 840
C		CBLL 850
165	CONTINUE	CBLL 860
	I2=I2+1	CBLL 870
	GO TO(166,168,170)I2	CBLL 880
166	I3=I3+1	CBLL 890
	DO 167 I=1,5	CBLL 900
167	PHIX(I3,I4+I)=DUMMY(I)	CBLL 910
	GO TO 150	CBLL 920
168	DO 169 I=1,5	CBLL 930
169	PHIY(I3,I4+I)=DUMMY(I)	CBLL 940
	GO TO 150	CBLL 950
170	DO 171 I=1,5	CBLL 960
171	PHIZ(I3,I4+I)=DUMMY(I)	CBLL 970
	I2=0	CBLL 980
	GO TO 150	CBLL 990
C		CBLL1000
C	SORT MODE SHAPE DATA	CBLL1010
C		CBLL1020
200	CONTINUE	CBLL1030
	NMN=5*NOSETS	CBLL1040
	DO 215 I=1,NNN	CBLL1050
	K=10	CBLL1060
	IF(I.EQ.MODE(1))K=1	CBLL1070
	IF(I.EQ.MODE(2))K=2	CBLL1080
	IF(I.EQ.MODE(3))K=3	CBLL1090
	IF(I.EQ.MODE(4))K=4	CBLL1100
	IF(I.EQ.MODE(5))K=5	CBLL1110

	IF(K.EQ.10)GO TO 215	CBLL1120
	DO 215 J=1,NOJTS	CBLL1130
	XPHI(J,K)=PHIX(J,I)	CBLL1140
	YPHI(J,K)=PHIY(J,I)	CBLL1150
	ZPHI(J,K)=PHIZ(J,I)	CBLL1160
	215 CONTINUE	CBLL1170
C		CBLL1180
C	PRINT DATA READ FROM TAPE4	CBLL1190
C		CBLL1200
	WRITE(6,401)	CBLL1210
	DO 220 I=1,NOJTS	CBLL1220
	220 WRITE(6,402)I,XJT(I),YJT(I),ZJT(I)	CBLL1230
	WRITE(6,403)	CBLL1240
	DO 225 J=1,NOMODE	CBLL1250
	WRITE(6,407)MODE(J)	CBLL1260
	DO 225 I=1,NOJTS	CBLL1270
	WRITE(6,402)I,XPHI(I,J),YPHI(I,J),ZPHI(I,J)	CBLL1280
	225 CONTINUE	CBLL1290
C		CBLL1300
C	DETERMINE C. G. LOCATION	CBLL1310
C		CBLL1320
	TOTM=0.0	CBLL1330
	XTOT=0.0	CBLL1340
	YTOT=0.0	CBLL1350
	ZTOT=0.0	CBLL1360
	DO 230 J=1,NOJTS	CBLL1370
	TOTM=TOTM+AMASS(J)	CBLL1380
	XTOT=XTOT+AMASS(J)*XJT(J)	CBLL1390
	YTOT=YTOT+AMASS(J)*YJT(J)	CBLL1400
	230 ZTOT=ZTOT+AMASS(J)*ZJT(J)	CBLL1410
	XCG=XTOT/TOTM	CBLL1420
	YCG=YTOT/TOTM	CBLL1430
	ZCG=ZTOT/TOTM	CBLL1440
	DO 235 J=1,NOJTS	CBLL1450
	XJT(J)=XJT(J)-XCG	CBLL1460
	YJT(J)=YJT(J)-YCG	CBLL1470
	235 ZJT(J)=ZJT(J)-ZCG	CBLL1480
C		CBLL1490
C	READ TIME HISTORY DATA FROM TAPE3	CBLL1500
C		CBLL1510
	REWIND 3	CBLL1520
	NOREAD=14+9*NOLEG+3*NOMODE	CBLL1530
	IJ=0	CBLL1540
	700 CONTINUE	CBLL1550
	IJ=IJ+1	CBLL1560
	IF(IJ.GT.NOTIME)STOP	CBLL1570
	710 READ(3)(DUMMY(I),I=1,NOREAD)	CBLL1580
	IF(ABS(DUMMY(1)-TYM(IJ)).GT.(DUMMY(2)/10.)) GO TO 710	CBLL1590
C		CBLL1600
C	TIME POINT HAS BEEN FOUND	CBLL1610
C		CBLL1620
	TYM(IJ)=DUMMY(1)	CBLL1630
	XSDD = DUMMY(3)	CBLL1640
	YSDD = DUMMY(4)	CBLL1650
	ZSDD =DUMMY(5)	CBLL1660
	PHI =DUMMY(6)	CBLL1670

	THTA =DUMMY(7)	CBLL1680
	PSI =DUMMY(8)	CBLL1690
	WX =DUMMY(9)	CBLL1700
	WY =DUMMY(10)	CBLL1710
	WZ =DUMMY(11)	CBLL1720
	WXD =DUMMY(12)	CBLL1730
	WYD =DUMMY(13)	CBLL1740
	WZD =DUMMY(14)	CBLL1750
	II=14	CBLL1760
	DO 701 I=1,NOMODE	CBLL1770
	I1=II+1+3*(I-1)	CBLL1780
	GC(I)=DUMMY(I1)	CBLL1790
	GCD(I)=DUMMY(I1+1)	CBLL1800
701	GCDD(I)=DUMMY(I1+2)	CBLL1810
	II=14+3*NOMODE	CBLL1820
	DO 702 I=1,NOLEG	CBLL1830
	I1=II+1+3*(I-1)	CBLL1840
	FMSX(I)=DUMMY(I1)	CBLL1850
	FMSY(I)=DUMMY(I1+1)	CBLL1860
702	FMSZ(I)=DUMMY(I1+2)	CBLL1870
	II=14+3*NOMODE+3*NOLEG	CBLL1880
	DO 703 I=1,NODS	CBLL1890
	I1=II+1+3*(I-1)	CBLL1900
	FDSX(I)=DUMMY(I1)	CBLL1910
	FDSY(I)=DUMMY(I1+1)	CBLL1920
703	FDSZ(I)=DUMMY(I1+2)	CBLL1930
	WRITE(6,619)	CBLL1940
	WRITE(6,614)	CBLL1950
	WRITE(6,610)TYM(IJ),XSDD,YSDD,ZSDD,PHI,THTA,PSI,WX,WY,WZ,	CBLL1960
	* WXD,WYD,WZD	CBLL1970
	DO 704 I=1,NOMODE	CBLL1980
704	WRITE(6,611)I,GC(I),GCD(I),GCDD(I)	CBLL1990
	WRITE(6,620)	CBLL2000
	DO 705 I=1,NOLEG	CBLL2010
705	WRITE(6,612)I,FMSX(I),FMSY(I),FMSZ(I)	CBLL2020
	WRITE(6,616)	CBLL2030
	DO 706 I=1,NODS	CBLL2040
706	WRITE(6,613)I,FDSX(I),FDSY(I),FDSZ(I)	CBLL2050
C		CBLL2060
C	CALCULATE DIRECTION COSINE MATRIX	CBLL2070
C		CBLL2080
	COSPFI=COS(PHI)	CBLL2090
	SINPHI=SIN(PHI)	CBLL2100
	COSTHA=COS(THTA)	CBLL2110
	SINTHA=SIN(THTA)	CBLL2120
	COSPSI=COS(PSI)	CBLL2130
	SINPSI=SIN(PSI)	CBLL2140
	DC(1,1)=COSTHA*COSPFI	CBLL2150
	A=SINTHA*COSPSI	CBLL2160
	B=COSPFI*SINPSI	CBLL2170
	DC(1,2)=SINPHI*A-B	CBLL2180
	C=SINPHI*SINPSI	CBLL2190
	DC(1,3)=COSPFI*A+C	CBLL2200
	DC(2,1)=COSTHA*SINPSI	CBLL2210
	DC(2,2)=SINTHA*C+COSPFI*COSPSI	CBLL2220
	DC(2,3)=SINTHA*B-SINPHI*COSPSI	CBLL2230

	DC(3,1)=-SINTHA	CBLL2240
	DC(3,2)=SINPHI*COSTHA	CBLL2250
	DC(3,3)=COSPHI*COSTHA	CBLL2260
	ZETA=-ZETA	CBLL2270
C		CBLL2280
C	ACCELERATION COMPONENTS	CBLL2290
C		CBLL2300
	GX=-GRAV*COS(ZETA)	CBLL2310
	GZ=GRAV*SIN(ZETA)	CBLL2320
	GRAVX=DC(1,1)*GX+DC(3,1)*GZ	CBLL2330
	GRAVY=DC(1,2)*GX+DC(3,2)*GZ	CBLL2340
	GRAVZ=DC(1,3)*GX+DC(3,3)*GZ	CBLL2350
	XCGA=DC(1,1)*XSDD+DC(2,1)*YSDD+DC(3,1)*ZSDD	CBLL2360
	YCGA=DC(1,2)*XSDD+DC(2,2)*YSDD+DC(3,2)*ZSDD	CBLL2370
	ZCGA=DC(1,3)*XSDD+DC(2,3)*YSDD+DC(3,3)*ZSDD	CBLL2380
	WXX=-(WY**2+WZ**2)	CBLL2390
	WXY=WX*WY	CBLL2400
	WXZ=WX*WZ	CBLL2410
	WYY=-(WX**2+WZ**2)	CBLL2420
	WYZ=WY*WZ	CBLL2430
	WZZ=-(WX**2+WY**2)	CBLL2440
	XSUM=0.	CBLL2450
	YSUM=0.	CBLL2460
	ZSUM=0.	CBLL2470
	TXSUM=0.	CBLL2480
	TYSUM=0.	CBLL2490
	TZSUM=0.	CBLL2500
C		CBLL2510
C	DETERMINE JOINT ACCELERATIONS	CBLL2520
C		CBLL2530
	DO 750 J=1,NOJTS	CBLL2540
	ELX=XJT(J)	CBLL2550
	ELY=YJT(J)	CBLL2560
	ELZ=ZJT(J)	CBLL2570
	ELDX=0.0	CBLL2580
	ELDY=0.0	CBLL2590
	ELDZ=0.0	CBLL2600
	ELDDX=0.0	CBLL2610
	ELDDY=0.0	CBLL2620
	ELDDZ=0.0	CBLL2630
	DO 720 I=1,NOMODE	CBLL2640
	ELX=ELX+XPHI(J,I)*GC(I)	CBLL2650
	ELY=ELY+YPHI(J,I)*GC(I)	CBLL2660
	ELZ=ELZ+ZPHI(J,I)*GC(I)	CBLL2670
	ELDX=ELDX+XPHI(J,I)*GCD(I)	CBLL2680
	ELDY=ELDY+YPHI(J,I)*GCD(I)	CBLL2690
	ELDZ=ELDZ+ZPHI(J,I)*GCD(I)	CBLL2700
	ELDDX=ELDDX+XPHI(J,I)*GCDD(I)	CBLL2710
	ELDDY=ELDDY+YPHI(J,I)*GCDD(I)	CBLL2720
	ELDDZ=ELDDZ+ZPHI(J,I)*GCDD(I)	CBLL2730
720	CONTINUE	CBLL2740
	SAX(J)=ELX	CBLL2750
	SAY(J)=ELY	CBLL2760
	SAZ(J)=ELZ	CBLL2770
	ACLX = XCGA+ELDDX-WZD*ELY+WYD*ELZ+2.*(-WZ*ELDY+WY*ELDZ)	CBLL2780
	* +WXX*ELX+WXY*ELY+WYZ*ELZ	CBLL2790

	ACLY = YCGA+ELDDY+WZD*ELX-WXD*ELZ+2.*(WZ*ELDX-WX*ELDZ)	CBLL2800
	* +WXY*ELX+WYY*ELY+WYZ*ELZ	CBLL2810
	ACLZ = ZCGA+ELDDZ-WYD*ELX+WXD*ELY+2.*(-WY*ELDX+WX*ELDY)	CBLL2820
	* +WXZ*ELX+WYZ*ELY+WZZ*ELZ	CBLL2830
	FX(J)=-ACLX*AMASS(J)	CBLL2840
	FY(J)=-ACLY*AMASS(J)	CBLL2850
	FZ(J)=-ACLZ*AMASS(J)	CBLL2860
750	CONTINUE	CBLL2870
C		CBLL2880
C	SUM FORCES AT EACH JOINT	CBLL2890
C		CBLL2900
	WRITE(6,615)	CBLL2910
	DO 755 J=1,NOJTS	CBLL2920
	FXJT(J)=0.	CBLL2930
	FYJT(J)=0.	CBLL2940
755	FZJT(J)=0.	CBLL2950
	DO 760 I=1,NOLEG	CBLL2960
	J=JOTMS(I)	CBLL2970
	FXJT(J)=FMSX(I)	CBLL2980
	FYJT(J)=FMSY(I)	CBLL2990
760	FZJT(J)=FMSZ(I)	CBLL3000
	DO 765 I=1,NODS	CBLL3010
	J=JOTDS(I)	CBLL3020
	FXJT(J)=FDSX(I)	CBLL3030
	FYJT(J)=FDSY(I)	CBLL3040
765	FZJT(J)=FDSZ(I)	CBLL3050
	DO 780 J=1,NOJTS	CBLL3060
	FXJT(J)=FXJT(J)+FX(J)+AMASS(J)*GRAVX	CBLL3070
	FYJT(J)=FYJT(J)+FY(J)+AMASS(J)*GRAVY	CBLL3080
	FZJT(J)=FZJT(J)+FZ(J)+AMASS(J)*GRAVZ	CBLL3090
C		CBLL3100
C	SUM FORCES AND MOMENTS AT C. G.	CBLL3110
C		CBLL3120
	XSUM=XSUM+FXJT(J)	CBLL3130
	YSUM=YSUM+FYJT(J)	CBLL3140
	ZSUM=ZSUM+FZJT(J)	CBLL3150
	TXSUM=TXSUM+SAY(J)*FZJT(J)-SAZ(J)*FYJT(J)	CBLL3160
	TYSUM=TYSUM+SAZ(J)*FXJT(J)-SAX(J)*FZJT(J)	CBLL3170
	TZSUM=TZSUM+SAX(J)*FYJT(J)-SAY(J)*FXJT(J)	CBLL3180
	WRITE(6,402)J,FXJT(J),FYJT(J),FZJT(J)	CBLL3190
780	CONTINUE	CBLL3200
C		CBLL3210
C	SUMMARY INFORMATION	CBLL3220
C		CBLL3230
	WRITE(6,617)XSUM,YSUM,ZSUM,TXSUM,TYSUM,TZSUM	CBLL3240
	GO TO 700	CBLL3250
C		CBLL3260
C	FORMAT STATEMENTS	CBLL3270
C		CBLL3280
400	FORMAT(5X,I3,2X,7E10.3)	CBLL3290
401	FORMAT(/15X,17HJOINT COORDINATES//4X,5HJOINT,7X,1HX,11X,1HY,11X,	CBLL3300
	* 1HZ)	CBLL3310
402	FORMAT(6X,I2,1X,3(2X,E10.3))	CBLL3320
403	FORMAT(/,15X,11HMODE SHAPES/)	CBLL3330
404	FORMAT(/10X,45HINCORRECT INFORMATION READ FROM TAPE4 - STOP)	CBLL3340
407	FORMAT(/6X,7HMODE = ,I2,/,4X,5HJOINT,7X,1HX,11X,1HY,11X,1HZ)	CBLL3350

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500 FORMAT(15I5) CBLL3360
501 FORMAT(5E10.3) CBLL3370
600 FORMAT(1H1,51X,33HCENTER BODY LANDING LOADS PROGRAM CBLL3380
* /42X,53HMASTER AGREEMENT, CBLL3390
*ONTRACT NAS1-8137, TASK ORDER FIVE/46X,45HMCDONNELL DOUGLAS ASTRONCBLL3400
*AUTICS COMPANY - EAST,////) CBLL3410
601 FORMAT(10X, 9HNOJTS = ,I10/10X,9HNOLEG = ,I10/10X,9HNOMODE = , CBLL3420
* I10/10X,9HNOTIME = ,I10/10X,9HNOSETS = ,I10) CBLL3430
602 FORMAT(10X,9HMODE = ,5I10) CBLL3440
603 FORMAT(10X,9HJOTMS = ,5I10) CBLL3450
604 FORMAT(10X,9HJOTDS = ,10I10) CBLL3460
605 FORMAT(10X,9HZETA = ,E10.3/10X,9HGRAV = ,E10.3) CBLL3470
606 FORMAT(10X,9HMASS = ,10E10.3,/(19X,10E10.3)) CBLL3480
607 FORMAT(10X,9HTYM = ,10E10.3/29X,10E10.3) CBLL3490
610 FORMAT(15X ,7HTIME = ,E10.3/10X,7HXSDD = ,E10.3,8H YSDD = , CBLL3500
* E10.3,8H ZSDD = ,E10.3/10X,7HPhi = ,E10.3,8H THTA = , CBLL3510
* E10.3,8H PSI = ,E10.3/10X,7HWX = ,E10.3,8H WY = , CBLL3520
* E10.3,8H WZ = ,E10.3/10X,7HWXD = ,E10.3,8H WYD = , CBLL3530
* E10.3,8H WZD = ,E10.3) CBLL3540
611 FORMAT(10X,7HMODE = ,I5,5X,8H GC = ,E10.3,8H GCD = ,E10.3, CBLL3550
* 8H GCDD = ,E10.3) CBLL3560
612 FORMAT(10X,7HI = ,I5,5X,8H FMSX = ,E10.3,8H FMSY = ,E10.3, CBLL3570
* 8H FMSZ = ,E10.3) CBLL3580
613 FORMAT(10X,7HI = ,I5,5X,8H FDSX = ,E10.3,8H FDSY = ,E10.3, CBLL3590
* 8H FDSZ = ,E10.3) CBLL3600
614 FORMAT(/,15X,17HTIME HISTORY DATA,/) CBLL3610
615 FORMAT(/,15X,38HCENTER BODY LANDING LOADS DISTRIBUTION//4X,5HJOINTCBLL3620
* ,7X,1HX,11X,1HY,11X,1HZ) CBLL3630
616 FORMAT(/,15X,16HDRAG STRUT LOADS,/) CBLL3640
617 FORMAT(/,15X,28HLANDING LOADS SUMMED AT C.G.//,6X,5HFX = , CBLL3650
* E10.3/6X,5HFY = ,E10.3/6X,5HFZ = ,E10.3/6X,5HTX = , CBLL3660
* E10.3/6X,5HTY = ,E10.3/6X,5HTZ = ,E10.3) CBLL3670
619 FORMAT(1H1) CBLL3680
620 FORMAT(/,15X,16HMAIN STRUT LOADS,/) CBLL3690
END CBLL3700

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APPENDIX F

NUMERICAL INTEGRATION ROUTINE
LANDING LOADS AND MOTIONS PROGRAM

F.1 Introduction - The methods and procedures employed during the numerical integration of the equations of motion in the Landing Loads and Motions Program are described in the following paragraphs. These operations are performed in the subroutines RKCUT and INITUP. The sequence of program calls for these subroutines are shown in Figure 5-17 and subroutine listings are given in Appendix I.

F.2 Method - The second-order equations of motion in the Landing Loads and Motions Program are reduced to an equivalent set of simultaneous first-order equations. Each of these first order equations takes the general form of

$$\dot{y} = f(y,t) \text{ where } y(t_0) = y_0 \quad (\text{F-1})$$

At any step n in the integration of these equations with respect to time t , the quantities t_n , y_n , and $\dot{y}_n = f(y_n, t_n)$ are available. To compute y_{n+1} and thus $\dot{y}_{n+1} = f(y_{n+1}, t_{n+1})$, where $t_{n+1} = t_n + \Delta t$, and Δt (program variable HZ) is the integration interval, the following Runge-Kutta formulas are used:

$$\begin{aligned} y_A &= y_n + \frac{\Delta t}{2} \dot{y}_n, & \dot{q}_n &= \dot{y}_n, & \dot{y}_A &= f(y_A, t_n + \frac{\Delta t}{2}) \\ y_B &= y_n + \frac{\Delta t}{2} \dot{y}_A, & \dot{q}_A &= \dot{q}_n + 2\dot{y}_A, & \dot{y}_B &= f(y_B, t_n + \frac{\Delta t}{2}) \\ y_C &= y_n + \Delta t \dot{y}_B, & \dot{q}_B &= \dot{q}_A + 2\dot{y}_B, & \dot{y}_C &= f(y_C, t_n + \Delta t) \\ y_{n+1} &= y_n + \Delta t (\dot{q}_B + \dot{y}_C), & \dot{q}_C &= \dot{y}_C, & \dot{y}_{n+1} &= f(y_{n+1}, t_n + \Delta t) \end{aligned} \quad (\text{F-2})$$

References (F-1) and (F-2) present an explanation of these formulas and a discussion of the Runge-Kutta method.

These formulas are applied to the system of equations using either a fixed or variable integration interval, Δt . When a variable Δt is used, the routine computes a truncation error indicator at each step of the integration, based on the quantity

$$E_{n+1} = \text{Max} \{ \text{Min} [|\Delta t (\dot{y}_{n+1}^i - \dot{y}_C^i)|, | \frac{\Delta t (\dot{y}_{n+1}^i - \dot{y}_C^i)}{y_{n+1}^i} |] \} \quad (\text{F-3})$$

where $i = 1, 2, \dots, N$ and N is the total number of integrated variables, and the superscript i denotes the i th integrated variable. The notation of Equation (F-3), indicates that for each i th integrated variable, the minimum of the two quantities

$$| \Delta t (\dot{y}_{n+1}^i - \dot{y}_C^i) | \quad \text{and} \quad | \frac{\Delta t (\dot{y}_{n+1}^i - \dot{y}_C^i)}{y_{n+1}^i} |$$

is saved. The truncation error indicator, E_{n+1} , is then set equal to the maximum of these minimums. The value of E_{n+1} is compared with the two constants E_{max} (EMAX) and E_{min} (EMIN) which are the input values for the maximum and minimum allowable truncation errors, respectively. This comparison results in a modification of Δt as follows:

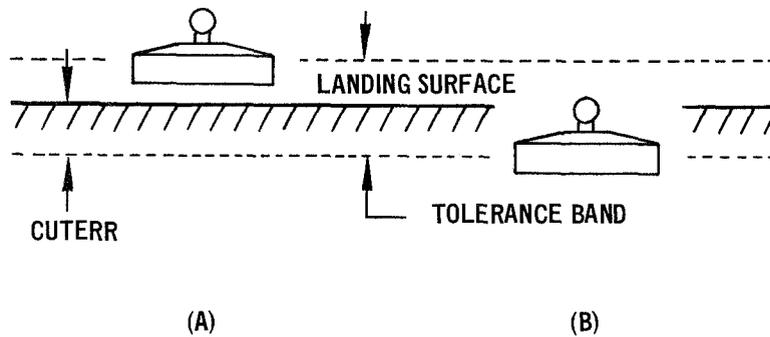
1. If $E_{n+1} < E_{\text{min}}$ for two consecutive integration steps, Δt is doubled and the integration continues with the new Δt . However, if Δt equals Δt_{max} (HMAX), Δt is not doubled.
2. If $E_{\text{min}} \leq E_{n+1} \leq E_{\text{max}}$, Δt is left unchanged.
3. If $E_{n+1} \geq E_{\text{max}}$, Δt is halved and the values at step n , which are saved in COMMON, are used to continue the integration. However, if Δt equals Δt_{min} (HMIN), Δt is not halved.

This procedure is continuously applied to the center body equations of motion, Equation (5-23). The equations of motion for the footpads, Equation (5-4), are continuously integrated by the above procedure when the "contacting footpad" option of the program is employed. If the "noncontacting footpad" option is specified, a footpad's equations of motion are integrated only when the footpad is determined to be in contact with the landing surface. For a footpad approaching the landing surface, contact is defined as the

condition when the bottom of the footpad is found to be within a tolerance band above and below the landing surface, as shown in Figure F-1(a). If this footpad is found to have penetrated below this tolerance band as shown in Figure F-1(b), a linear interpolation procedure is employed to estimate at what time the footpad would have entered the tolerance band. The integration is backed up and then continued from this point in time with the footpad considered contacting.

F.3 Subroutine Calls - There are six calls to the integration routine. A brief description of each of the entries into the subroutines is given below.

<u>Subroutine</u>	<u>Entry Point</u>	<u>Operation</u>
INITUP	LOC	Sets up a list of cutoff variables and their cutoff values.
INITUP	INUPD	Sets up a list of all integrated variables.
RKCUT	SETUP	Initializes numerical integration procedure for type of integration requested.
RKCUT	INTEG	Performs all integration and computes truncation error indicator.
RKCUT	UPDATE	Updates integration variables and modifies integration interval when required.
RKCUT	CUT	Checks cutoff limits for introduction of footpad equations into integration routine.



**FIGURE F-1 INTEGRATION CONTROL TOLERANCE FOR
FOOTPAD EQUATIONS OF MOTION**

F-4 References

- F-1 Hamming, R. W., Numerical Methods for Scientists and Engineers, McGraw-Hill Book Company, New York, 1962, pp. 212-213.
- F-2 Conte, S. D., Elementary Numerical Analysis, McGraw-Hill Book Company, New York, 1965, pp. 204-258.

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APPENDIX G

CENTER BODY INTERNAL LOAD
DISTRIBUTION FOR TASK ORDER THREE
LANDER USING REFINED IDEALIZATION

A refined idealization of the Task Order Three lander center body, shown in Figure G-1, was employed to determine displacements and internal loads for the loading condition defined in Section 6.2.4. This idealization, which employs 72 joints and 108 bar elements, includes a more refined model of the side beams and radial beams. The center hub was modeled with nine elements to more nearly represent a cylinder. In addition, clevis fittings, which connect the drag struts of a gear to the center body, were idealized with elements such as 4, 11, 29, and 36. Accordingly, joints 22 and 27 (and 40 and 45 which are hidden) are the actual center body attach points for gears 2 and 3 and were assumed to be pinned supports. Main strut attach points (joints 25 and 43) for gears 2 and 3 were also assumed to be pinned supports.

Loads that were applied to the joints in the idealization of Figure 6-8 were applied to coincident joints in Figure G-1. For example, loads that were applied to joint 6 in Figure 6-8 were applied to joint 10 in the refined idealization. Loads that were applied to joints 31, 32 and 33 (on the center hub) in Figure 6-8 were distributed to joints 67 through 72, some of which are hidden in Figure G-1. These loads were distributed to the refined idealization joint locations in a statically equivalent manner.

A comparison of selected displacements and internal loads obtained with the 33 joint idealization and the refined idealization is given in Figure G-2.

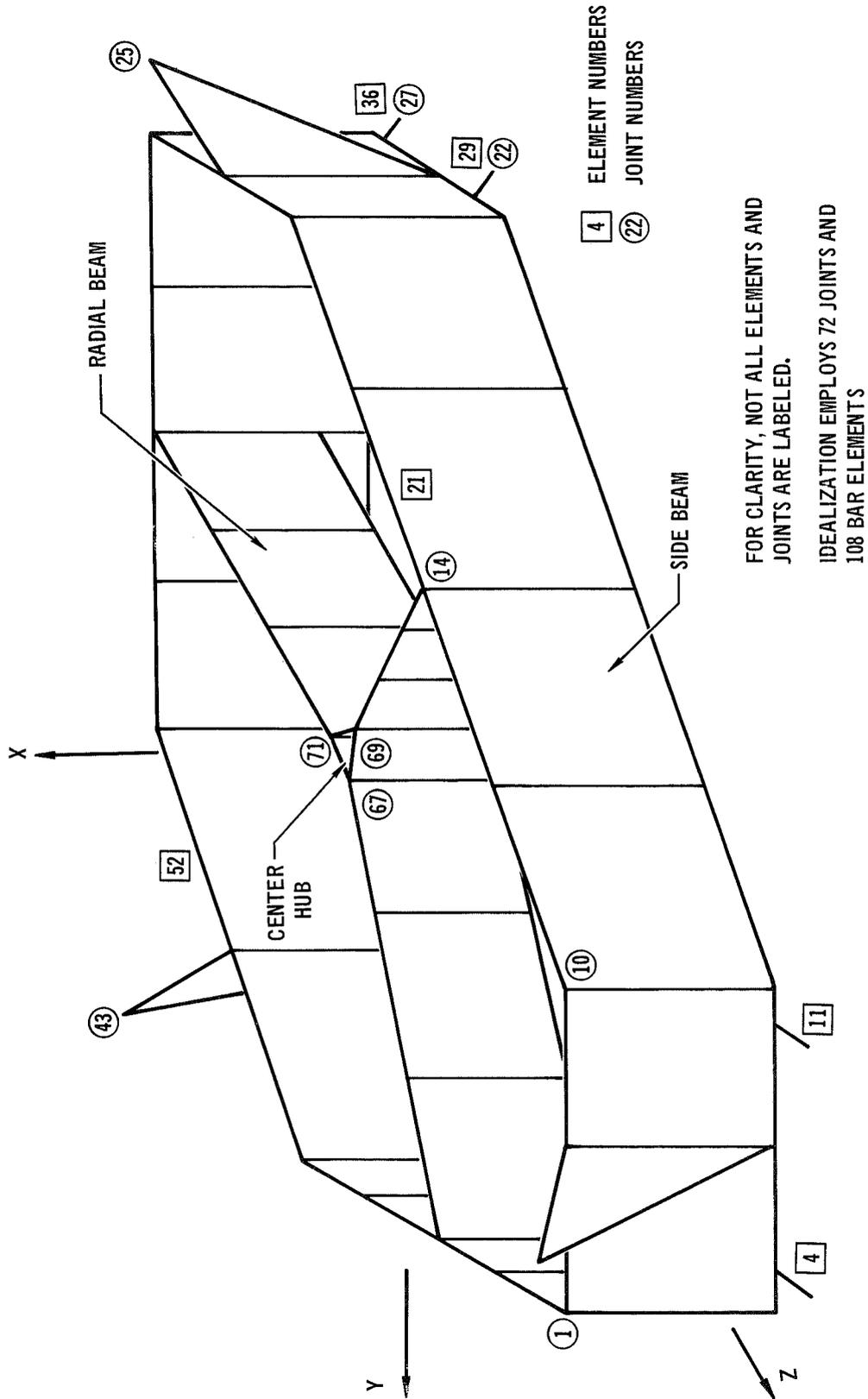


FIGURE G-1 REFINED CENTER BODY STRUCTURAL IDEALIZATION

	JOINT DISPLACEMENTS (CM)		
	X	Y	Z
JOINT 1 (FIGURE 6-8)*	-2.594×10^{-1}	-1.284×10^{-5}	9.955×10^{-2}
JOINT 1 (FIGURE G-1)**	-5.325×10^{-1}	-1.877×10^{-4}	1.775×10^{-1}
JOINT 8 (FIGURE 6-8)	-1.342×10^{-1}	2.195×10^{-1}	-2.636×10^{-2}
JOINT 14 (FIGURE G-1)	-2.365×10^{-1}	4.810×10^{-1}	-1.007×10^{-1}

*FIGURE 6-8 ILLUSTRATES THE 33 JOINT IDEALIZATION.

**FIGURE G-1 ILLUSTRATES THE REFINED (72 JOINT) IDEALIZATION.

	ELEMENT INTERNAL LOADS AT "P" END					
	FORCE X (DYNES)	FORCE Y (DYNES)	FORCE Z (DYNES)	MOMENT X (CM-DYNES)	MOMENT Y (CM-DYNES)	MOMENT Z (CM-DYNES)
ELEMENT 14 (FIGURE 6-8)	-2.588×10^8	9.057×10^7	-1.025×10^8	9.972×10^6	5.477×10^9	4.688×10^9
ELEMENT 21 (FIGURE G-1)	-2.140×10^8	1.230×10^8	-1.778×10^8	1.088×10^7	3.830×10^9	8.550×10^9
ELEMENT 32 (FIGURE 6-8)	-5.394×10^8	6.205×10^8	1.633×10^8	5.697×10^6	-4.800×10^9	1.061×10^{10}
ELEMENT 52 (FIGURE G-1)	-1.115×10^9	1.163×10^9	1.822×10^8	5.800×10^7	-6.190×10^9	1.756×10^{10}

FIGURE G-2 COMPARISON OF COMPUTER RESULTS FOR 33 AND 72 JOINT IDEALIZATIONS OF TASK ORDER THREE LANDER CENTER BODY

APPENDIX H

PROGRAM LISTING

STRUCTURAL ANALYSIS PROGRAM

OVERLAY (SASLP, 0, 0)	OV00	10
PROGRAM SAPT5 (INPUT,OUTPUT,TAPE5=INPUT,TAPE6=OUTPUT,	MAIN	10
1 TAPE1,TAPE2,TAPE9)	MAIN	20
COMMON COM(30)	MAIN	30
EQUIVALENCE (COM(17), INDRKT)	MAIN	40
EQUIVALENCE (COM(24), INDNMA)	MAIN	50
EQUIVALENCE (COM(30), INDPGM)	MAIN	60
C	MAIN	70
C THE DIMENSION OF COMAIN CAN BE INCREASED OR DECREASED	MAIN	80
C TO CONTROL THE STORAGE AVAILABLE FOR THE STIFFNESS	MAIN	90
C MATRIX (IF ISFDIM IS ALSO CHANGED).	MAIN	100
C	MAIN	110
C COMMON / CMAIN / COMAIN(14455)	MAIN	120
C COMMON / CMAIN / COMAIN(16955)	MAIN	130
5 CONTINUE	MAIN	140
C INITIALIZE WRSTRK ROUTINE	MAIN	150
C CALL WRSTR1(I,J,DUM)	MAIN	160
C	MAIN	170
C DATA AND VARIABLE SET-UP	MAIN	180
C	MAIN	190
C CALL SECOND (TIME0)	MAIN	200
C CALL OVERLAY (5LSASLP,1,0,6HRECALL)	MAIN	210
C CALL SECOND (TIME1)	MAIN	220
C IF (INDPGM .EQ. 0) GO TO 8	MAIN	230
C	MAIN	240
C GEAR ANALYSIS INPUT AND PROCESSING ROUTINES	MAIN	250
C	MAIN	260
C CALL OVERLAY (6LGEARRT,5,0,6HRECALL)	MAIN	270
C CALL SECOND(TIME2)	MAIN	280
C GO TO 20	MAIN	290
8 CONTINUE	MAIN	300
C	MAIN	310
C CENTER BODY OPTION ROUTINES	MAIN	315
C LOCAL STIFFNESS AND TRANSFORMATION MATRICES AND STRUCTURAL	MAIN	320
C STIFFNESS MATRIX GENERATION	MAIN	330
C	MAIN	340
C IF (INDRKT.EQ.0) CALL OVERLAY (5LSASLP,2,0,6HRECALL)	MAIN	350
C CALL SECOND (TIME2)	MAIN	360
C *****	MAIN	370
C * OVERLAYS THREE AND FOUR ARE MUTUALLY EXCLUSIVE *	MAIN	380
C *****	MAIN	390
C IF (INDNMA.NE.0) GO TO 10	MAIN	400
C	MAIN	410
C DISPLACEMENT, ROTATION, FORCE, AND MOMENT SOLUTION	MAIN	420
C	MAIN	430
C CALL OVERLAY (5LSASLP,3,0,6HRECALL)	MAIN	440
C CALL SECOND (TIME3)	MAIN	450
C GO TO 20	MAIN	460
C	MAIN	470
C NORMAL MODE ANALYSIS SECTION	MAIN	480
C	MAIN	490
10 CALL OVERLAY (5LSASLP,4,0,6HRECALL)	MAIN	500
C CALL SECOND (TIME4)	MAIN	510
20 CONTINUE	MAIN	520

	T01=TIME1-TIME0	MAIN 530
	T12=TIME2-TIME1	MAIN 540
	WRITE(6,40)TIME0,TIME1,T01	MAIN 550
	IF (INDPGM .EQ. 0) GO TO 25	MAIN 560
	WRITE(6,44)TIME1,TIME2,T12	MAIN 570
	GO TO 5	MAIN 580
25	CONTINUE	MAIN 590
	WRITE(6,42)TIME1,TIME2,T12	MAIN 600
	IF (INDNMA.EQ.0) GO TO 30	MAIN 610
	T23OR4=TIME4-TIME2	MAIN 620
	WRITE (6,50) TIME2,TIME4,T23OR4	MAIN 630
	GO TO 5	MAIN 640
30	CONTINUE	MAIN 650
	T23OR4=TIME3-TIME2	MAIN 660
	WRITE (6,60) TIME2,TIME3,T23OR4	MAIN 670
	GO TO 5	MAIN 680
C		MAIN 690
40	FORMAT (1H138X,26HCPU TIME USAGE TABLE (SEC)///63X32H TIME IN	MAIN 700
	1TIME OUT TOTAL,/33H0INPUT AND INITIALIZATION OVERLAY,26X2(2XF	MAIN 710
	2 10.3),1XF10.2 /)	MAIN 720
42	FORMAT (47H0STRUCTURAL STIFFNESS MATRIX GENERATION OVERLAY,	MAIN 730
	1 14XF10.3, 2XF10.3, 1XF10.2 /)	MAIN 740
44	FORMAT (32H0GEAR INPUT AND ANALYSIS OVERLAY, 27X2(2XF10.3),	MAIN 750
	1 1XF10.2)	MAIN 760
50	FORMAT (29H0NORMAL MODE ANALYSIS OVERLAY,32XF10.3,2XF10.3,1XF10.2)MAIN	MAIN 770
60	FORMAT (59H0DISPLACEMENT, ROTATION, FORCE, AND MOMENT SOLUTION OVE	MAIN 780
	1RLAY,2XF10.3,2XF10.3,1XF10.2//)	MAIN 790
	END	MAIN 800

	SUBROUTINE ERPNT1 (ALPHA,LENGTH,ICODE)	ERPNT1	10
C	PRINT ERROR MESSAGES	ERPNT1	20
C	ICODE = 0, NON-FATAL ERROR, CONTINUE	ERPNT1	30
C	ICODE = 1, FATAL ERROR, STOP	ERPNT1	40
C	ICODE =-1, FATAL ERROR, SET FATAL ERROR INDICATOR AND	ERPNT1	50
C	CONTINUE	ERPNT1	60
	DATA IFATAL / 0 /	ERPNT1	70
	DIMENSION ALPHA(13)	ERPNT1	80
	IF(ICODE.NE.0) GO TO 70	ERPNT1	82
	WRITE(6,50) (ALPHA(I),I=1,LENGTH)	ERPNT1	84
	GO TO 30	ERPNT1	86
70	CONTINUE	ERPNT1	88
	WRITE (6,40) (ALPHA(I),I=1,LENGTH)	ERPNT1	90
	IF (ICODE) 10,30,20	ERPNT1	100
10	IFATAL=1	ERPNT1	110
	GO TO 30	ERPNT1	120
20	STOP	ERPNT1	130
30	RETURN	ERPNT1	140
C	CHECK ENTRY FOR FATAL INDICATOR	ERPNT1	150
	ENTRY ERPNT2	ERPNT1	160
	IF (IFATAL.EQ.1) STOP	ERPNT1	170
	RETURN	ERPNT1	180
C		ERPNT1	190
	40 FORMAT(20H ***ERROR*** /13A10//)	ERPNT1	200
	50 FORMAT(20H ***WARNING*** /13A10//)	ERPNT1	205
	END	ERPNT1	210

. SUBROUTINE WRSTRK (IROW,JCOL,STFNZ)	WRST 10
C	WRST 20
C IROW = ROW NO. JCOL = COLUMN NO.	WRST 30
C STFNZ = VARIABLE	WRST 40
C	WRST 50
DIMENSION JCOLD(6), STFNZD(6), COL(6)	WRST 60
DATA COL / 6*4HCOL. /	WRST 70
C	WRST 80
C THE NON ZERO ELEMENTS OF THE STRUCTURAL STIFFNESS	WRST 90
C MATRIX ARE PASSED ONE AT A TIME AND PRINTED BY ROWS SIX	WRST 100
C VALUES PER LINE	WRST 110
C IF (IRSAVE.EQ.IROW) GO TO 10	WRST 120
C	WRST 130
C IF NEW ROW, PRINT ANY VARIABLES OF THE LAST ROW BEING HELD IN	WRST 140
C A LINE AND START A NEW LINE	WRST 150
C INDRNS=1	WRST 160
C GO TO 20	WRST 170
10 IF (ICOUNT.NE.6) GO TO 40	WRST 180
C	WRST 190
C IF SIX VARIABLES HAVE BEEN STORED IN A LINE, PRINT IT AND	WRST 200
C START A NEW LINE.	WRST 210
20 CONTINUE	WRST 220
IF (INDLIN.NE.1) GO TO 50	WRST 230
WRITE (6,80) IRSAVE,(COL(I),JCOLD(I),STFNZD(I),I=1,ICOUNT)	WRST 240
LINET=LINET+2	WRST 250
INDLIN=2	WRST 260
30 ICOUNT=0	WRST 270
IF (INDRNS.EQ.0) GO TO 40	WRST 280
INDRNS=0	WRST 290
INDLIN=1	WRST 300
IRSAVE=IROW	WRST 310
40 ICOUNT=ICOUNT+1	WRST 320
JCOLD(ICOUNT)=JCOL	WRST 330
STFNZD(ICOUNT)=STFNZ	WRST 340
IF (LINET.GT.56) GO TO 60	WRST 350
RETURN	WRST 360
50 WRITE (6,90) (COL(I),JCOLD(I),STFNZD(I),I=1,ICOUNT)	WRST 370
LINET=LINET+1	WRST 380
GO TO 30	WRST 390
60 IF (IROW.LT.0) RETURN	WRST 400
WRITE (6,70)	WRST 410
LINET=3	WRST 420
RETURN	WRST 430
C	WRST 440
C INITIALIZE THIS ROUTINE	WRST 450
C	WRST 460
ENTRY WRSTR1	WRST 470
LINET = 59	WRST 480
ICOUNT = 0	WRST 490
INDLIN = 1	WRST 500
INDRNS = 0	WRST 510
IRSAVE = 1	WRST 520
RETURN	WRST 530
C	WRST 540
C FORMAT 100 IS USED TO PRINT THE FIRST LINE FOR EACH ROW	WRST 550

```
C          FORMAT 200 IS USED FOR ALL OTHER LINES ( INDLIN .NE. 1)      WRST 560
C                                                                 WRST 570
C                                                                 WRST 580
70  FORMAT (28H1STRUCTURAL STIFFNESS MATRIX//)                          WRST 590
80  FORMAT (5H0ROW I3,6(1XA4,I4,1H=E10.3))                             WRST 600
90  FORMAT (8X,6(1XA4,I4,1H=E10.3))                                     WRST 610
END                                                                 WRST 620
```

	OVERLAP (SASLP, 1, 0)	OV10 10
	PROGRAM INITIAL	INIT 10
	COMMON COM(30)	INIT 20
	EQUIVALENCE (COM(30), INDPGM)	INIT 30
C		INIT 40
C	READ DATA CARDS AND SORT INTO PROPER ORDER	INIT 50
C		INIT 60
	CALL DATSET	INIT 70
	IF(INDPGM.EQ.1)RETURN	INIT 80
C		INIT 90
C	READ VARIABLE INPUT VALUES ARRANGED BY (DATSET)	INIT 100
C		INIT 110
	CALL RDDATA	INIT 120
	RETURN	INIT 130
	END	INIT 140

SUBROUTINE DATSET	DATS	10
DIMENSION CARD(8), IDATA(7)	DATS	20
COMMON COM(30)	DATS	30
EQUIVALENCE (COM(1), NJOINT)	DATS	40
EQUIVALENCE (COM(2), NFORCE)	DATS	50
EQUIVALENCE (COM(3), NMOMNT)	DATS	60
EQUIVALENCE (COM(4), NBAR)	DATS	70
EQUIVALENCE (COM(5), NJPNT)	DATS	80
EQUIVALENCE (COM(20), NLIMIT)	DATS	90
EQUIVALENCE (COM (29), NSHPAN)	DATS	100
EQUIVALENCE (COM(30), INDPGM)	DATS	110
DIMENSION ERL(5)	DATS	120
DIMENSION ERJ(5), ERF(5), ERM(5), ERB(5)	DATS	130
DIMENSION ERSH(5)	DATS	140
DATA ERSR /10H THE SHEAR, 10H PANELS AR, 10HE NOT NUMB,	DATS	150
1 10HERED SEQUE, 10HNTIALLY. /	DATS	160
DATA ERL / 10H THE LIMIT, 10HS ARE NOT, 10H NUMBERED ,	DATS	170
1 10HSEQUENTIAL, 10HLY /	DATS	180
DATA ERJ / 10H THE NODAL, 10H POINTS AR, 10HE NOT NUMB,	DATS	190
1 10HERED SEQUE, 10HNTIALLY /	DATS	200
DATA ERF / 10H THE FORCE, 10H VECTORS A, 10HRE NOT NUM,	DATS	210
1 10HBERED SEQU, 10HENTIALLY /	DATS	220
DATA ERM / 10H THE MOMEN, 10HT VECTORS , 10HARE NOT NU,	DATS	230
1 10HMBERED SEQ, 10HENTIALLY /	DATS	240
DATA ERB / 10H THE BAR D, 10HEFINITIONS, 10H ARE NOT N,	DATS	250
1 10HUMBERED SE, 10HQUENTIALLY/	DATS	260
DIMENSION EDHDC(6)	DATS	270
DATA GEAR / 10HGEAR /	DATS	280
DATA STRUCT / 10HSTRUCTURE /	DATS	290
DATA EDHDC / 10H THE DATA , 10HCASE DOES , 10HNOT HAVE A,	DATS	300
1 10HN ACCEPTAB, 10HLE HEADER , 10HCARD. /	DATS	310
NJOINT=0	DATS	320
NLIMIT=0	DATS	330
NFORCE=0	DATS	340
NMOMNT=0	DATS	350
NBAR=0	DATS	360
NJPNT=0	DATS	370
NSHPAN = 0	DATS	380
REWIND 1	DATS	390
MJOINT = 0	DATS	400
MJPNT = 0	DATS	410
MLIMIT = 0	DATS	420
MFORCE = 0	DATS	430
MMOMNT = 0	DATS	440
MBAR = 0	DATS	450
MSHPAN = 0	DATS	460
IFRIST = 0	DATS	470
IBADCD = 0	DATS	480
IFRST = 0	DATS	490
10 READ (5,200) CARD	DATS	500
IF (FOF, 5) 12, 14	DATS	510
12 WRITE(6, 240)	DATS	520
240 FORMAT (39H1END OF JOB---END OF DATA SET ON UNIT 5)	DATS	530
STOP	DATS	540
14 CONTINUE	DATS	550

	IF (IFRST .NE. 0) GO TO 17	DATS 560
	WRITE (6,190)	DATS 570
	IFRST = 1	DATS 580
17	CONTINUE	DATS 590
	IF (IFRIST .NE. 0) GO TO 16	DATS 600
C		DATS 610
C	CHECK FOR DATA SET HEADER CARD (GEAR) OR (STRUCTURE)	DATS 620
C	LEFT JUSTIFIED IN COL. 1-10	DATS 630
C		DATS 640
	INDPGM = 1	DATS 650
	IF (CARD(1) .EQ. GEAR) GO TO 13	DATS 660
	INDPGM = 0	DATS 670
	IF (CARD(1) .EQ. STRUCT) GO TO 15	DATS 680
	IF (IBADCD .EQ. 0) CALL ERPNT1(EDHDC, 6, 0)	DATS 690
	IBADCD = 1	DATS 700
	WRITE(6,230) CARD	DATS 710
	GO TO 10	DATS 720
13	CONTINUE	DATS 730
	IF (IBADCD .NE. 0) WRITE(6,190)	DATS 740
	RETURN	DATS 750
15	CONTINUE	DATS 760
	IF (IBADCD .NE. 0) WRITE(6,190)	DATS 770
	WRITE(6,192)	DATS 780
	IFRIST = 1	DATS 790
	GO TO 10	DATS 800
16	CONTINUE	DATS 810
	WRITE (6,230) CARD	DATS 820
C		DATS 830
C	CONVERT CODE AND SUBSCRIPT TO INTEGER	DATS 840
C		DATS 850
	DECODE (5,210,CARD(1)) ICODE, INDEX	DATS 860
C		DATS 870
C	ANY CODF NO. .GT. 7 IS CONSIDERED AN END OF RECORD	DATS 880
C	IF (ICODE.GT.7) GO TO 110	DATS 890
C		DATS 900
C	IF COLUMN ONE IS BLANK OR ZERO CONSIDER IT A COMMENT	DATS 910
C	IF (ICODE.EQ.0) GO TO 10	DATS 920
		DATS 930
	WRITE(1,300) ICODE	DATS 940
	WRITE(1,200) CARD	DATS 950
	GO TO (20,40,50,60,70,80,90), ICODE	DATS 960
20	NJOINT=NJOINT+1	DATS 970
	MJOINT = MAX0(MJOINT, INDEX)	DATS 980
	GO TO 10	DATS 990
40	NJPNT=NJPNT+1	DATS1000
	MJPNT = MAX0(MJPNT , INDEX)	DATS1010
	GO TO 10	DATS1020
50	NLIMIT=NLIMIT+1	DATS1030
	MLIMIT = MAX0(MLIMIT, INDEX)	DATS1040
	GO TO 10	DATS1050
60	NFORCE=NFORCE+1	DATS1060
	MFORCE = MAX0(MFORCE, INDEX)	DATS1070
	GO TO 10	DATS1080
70	NMOMNT=NMMOMNT+1	DATS1090
	MMOMNT = MAX0(MMOMNT, INDEX)	DATS1100
	GO TO 10	DATS1110

80	NBAR=NBAR+1	DATS1120
	MBAR = MAXO(MBAR , INDEX)	DATS1130
	GO TO 10	DATS1140
90	CONTINUE	DATS1150
	MSHPAN = MAXO(MSHPAN, INDEX)	DATS1160
	NSHPAN = NSHPAN + 1	DATS1170
	GO TO 10	DATS1180
110	CONTINUE	DATS1190
C	CHECK TO SEE IF INPUT CARDS ARE SEQUENCED PROPERLY	DATS1200
C	(THIS WILL NOT CATCH ALL KEY PUNCH ERRORS)	DATS1210
	NJPNT = NJPNT + NJOINT	DATS1220
	MJPNT = MAXO(MJPNT,MJOINT)	DATS1230
	IF (MJPNT .NE. NJPNT) CALL ERPNT1(ERJ, 5, -1)	DATS1240
	IF (MLIMIT .NE. NLIMIT) CALL ERPNT1(ERL, 5, -1)	DATS1250
	IF (MFORCE .NE. NFORCE) CALL ERPNT1(ERF, 5, -1)	DATS1260
	IF (MMOMNT .NE. NMOMNT) CALL ERPNT1(ERM, 5, -1)	DATS1270
	IF (MBAR .NE. NBAR) CALL ERPNT1(ERB, 5, -1)	DATS1280
	IF (MSHPAN .NE. NSHPAN) CALL ERPNT1(ERS, 5, -1)	DATS1290
C		DATS1300
C	IF A FATAL ERROR HAS OCCURRED, STOP	DATS1310
C		DATS1320
	CALL ERPNT2	DATS1330
C		DATS1340
	END FILE 1	DATS1350
	RETURN	DATS1360
C		DATS1370
190	FORMAT (1H1, 43X 45HSTRUCTURAL ANALYSIS PROGRAM --- LEGGED LANDER	DATS1380
	1/37X60HMASTER AGREEMENT, CONTRACT NAS1-8137, TASK ORDER NUMBER FIV	DATS1390
	2E/45X44HMC DONNELL DOUGLAS ASTRONAUTICS COMPANY, EAST	DATS1400
	3 ///)	DATS1410
192	FORMAT (DATS1420
	337H STRUCTURAL ANALYSIS DATA - CARD CODE, //28X 27HBLANK - 0	DATS1430
	4 COMMENTS ,/32X1H1,13X,23HNODAL POINT DEFINITIONS,/32X1H2,13X	DATS1440
	5,16HREFERENCE POINTS,/32X1H3,13X,33HNODAL POINT RESTRAINT DEFINIT	DATS1450
	6ONS,/32X1H4,13X,13HFORCE VECTORS,/32X1H5,13X,15HMOMENT VECTORS ,/3	DATS1460
	7 2X1H6, 13X, 15HBAR DEFINITIONS, / 32X1H7,13X	DATS1470
	8 23HSHEAR PANEL DEFINITIONS,/32X1H8,13X,24HFORMATED-DATA TERMINAT	DATS1480
	9OR /)	DATS1490
200	FORMAT (8A10)	DATS1500
210	FORMAT (I1, I4)	DATS1510
220	FORMAT (1X14)	DATS1520
230	FORMAT (1X8A10)	DATS1530
300	FORMAT(I4)	DATS1540
	END	DATS1550

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SUBROUTINE RDDATA
COMMON  NJOINT, NFORCE, NMOMNT, NBAR , NJPNT , ITINJO, IBAR
1      , NROW , INDSFL, INDSFG, IND1SL, IND1TR, ERR1TOL, RELAXF
2      , INDR1X, INDR1KT, INDR1PLS, MINRST, NLIMIT, NRWTOR
3      , IREDTO, NEIGVL, INDNMA, INDR1WNM, TMAX , ISFDIM, INDR1WTS
4      , NSHPAN, INDR1PGM
C      TOTAL LENGTH WITH STRSTF(0) .EQ. 4049
COMMON / CMAIN /
1      RJXYZ (100,3), IJRFM ( 74,6), IJALFM( 74,2), FRCVCT( 74,3),
2      RMTVCT( 74,3), IBARP ( 130), IBARQ ( 130), IBARR ( 130),
3      KS ( 130), BAREA ( 130), BARIN ( 130), BARIT ( 130),
4      BARJ ( 130), BARYM ( 130), BARSM ( 130), RKN ( 130),
5      RKT ( 130), SOLVEC( 444), RLIMTU( 88), RLIMTL( 88),
6      DPR1TIT( .88), IROWL ( 445), STRSTF(1)
COMMON/SP/IAPAN(30), IBPAN(30), ICPAN(30), IDPAN(30),
1      VSPAN(30), ESPAN(30), TSPAN(30)
DIMENSION STIFML(12,12), TRANSM(3,3), AMASS(102), IRWKP(103)
EQUIVALENC ( KS(1), STIFML(1,1)), ( BARIT(1), TRANSM(1,1))
EQUIVALENC ( RLIMTU(1),AMASS(1)), ( RLIMTL(15),IRWKP(1 ))
DIMENSION ERORD(4), ERRWKP(5)
DIMENSION ERLT(5)
DIMENSION ERRED(4)
DIMENSION ERFROW(5)
DIMENSION ERLN(5)
DIMENSION ERPQ(4)
DIMENSION ERBN(4), ERJNL(4)
DIMENSION ERJN( 5), ERJR( 7), ERFR( 6), ERMT(6)
DIMENSION EREGVL(8)
DATA EREGVL/ 10H AN EXCESS, 10HIVE NUMBER, 10H OF EIGENV,
1      10HALUES HAVE, 10H BEEN REQU, 10HESTED BY U,
2      10HSE OF NEIG, 10HVL. /
DATA ERRED / 40H IREDTO CAN NOT BE GREATER THAN NROW /
DATA ERORD /40H IREDTO CAN NOT BE LARGER THAN 102 /
DATA ERRWKP/50H THE IRWKP ARRAY IS NOT FILLED IN ASCENDING ORDER /
DATA ERPQ / 35H A ZERO LENGTH BAR IS NOT PERMITTED /
DATA ERBN/ 10H THERE MUS, 10HT BE AT LE, 10HAST DNE BA,
1      10HR /
DATA ERJNL/10H THERE MUS, 10HT BE AT LE, 10HAST TWO NO,
1      10H DAL POINTS/
DATA ERLN /50H A RESTRAINT INDICATOR IS OUT OF BOUNDS /
DATA ERFROW /45H INPUT DATA DISAGREES WITH MATRICES ON TAPE9 /
DATA ERLT /50H LOWER REACTION LIMIT CONFLICTS WITH UPPER LIMIT /
DATA ERJN /10H A BAR DEF,10HINITIATION US,10HES AN UNDE,10HFINED NODARRDDA 430
1      ,10HL POINT /
DATA ERJR /10H A BAR DEF,10HINITIATION US,10HES AN UNDE,10HFINED POINRRDDA 450
1      ,10HT TO DEFIN,10HE THE BEND,10HING PLANES /
DATA ERFR /10H A NODAL P,10HOINT IS AS,10HSOCIATED W,10HITH AN UNDRDDA 470
1      ,10HEFINED FOR,10HCE VECTOR /
DATA ERMT /10H A NODAL P,10HOINT IS AS,10HSOCIATED W,10HITH AN UNDRDDA 490
1      ,10HEFINED MOM,10HENT VECTOR/
DIMENSION ERSP3(3), ERSP4(4), ERSP5(5), ERSP6(4), ERSP7(4),
* ERSP8(4), ERSP9(4)
DATA ERSP3 / 10H ONLY 130 , 10HBARS ARE A, 10HLLLOWED. /
DATA ERSP4 / 10H ONLY 74 , 10HBAR NODE P, 10HOINTS ARE ,
1      10HALLLOWED. /

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DATA ERSP9 / 10H ONLY 30 , 10HSHEAR PANE, 10HLS ARE ALL, RDDA 560
1 10HOWED. / RDDA 570
DATA ERSP5 / 10H ONLY 26 , 10HREFERENCE , 10H NODE POIN, RDDA 580
1 10HTS ARE ALL, 10HOWED. / RDDA 590
DATA ERSP6 / 10H ONLY 88 , 10HLIMIT CARD, 10HS ARE ALLO, RDDA 600
1 10HWED. / RDDA 610
DATA ERSP7 / 10H ONLY 74 , 10HFORCE VECT, 10HORS ARE AL, RDDA 620
1 10HLOWED. / RDDA 630
DATA ERSP8 / 10H ONLY 74 , 10HMOMENT VEC, 10HTORS ARE A, RDDA 640
1 10HLOWED. / RDDA 650
NAMELIST / INDATA / INDSFL, INDSFG, IND1SL, INDITR, ERRTOL,SOLVEC RDDA 660
1 , RELAXF, INDRX RDDA 670
2 , INDRKT, INDWKT, INDPLS, MINRST RDDA 680
3 , INDNMA, IRWKP, AMASS, IREDTO, INDWNM, TMAX RDDA 690
4 , ISFDIM, INDWTS RDDA 700
* , NEIGVL RDDA 710
C RDDA 720
C CHECK INPUT DATA AGAINST VARIABLE DIMENSION LIMITS RDDA 730
IF ( NJOINT .GT. 74 ) CALL ERPNT1( ERSP4, 4, -1 ) RDDA 740
IF ( NJPNT .GT. 100 ) CALL ERPNT1( ERSP5, 5, -1 ) RDDA 750
IF ( NLIMIT .GT. 88 ) CALL ERPNT1( ERSP6, 4, -1 ) RDDA 760
IF ( NFORCE .GT. 74 ) CALL ERPNT1( ERSP7, 4, -1 ) RDDA 770
IF ( NMOMNT .GT. 74 ) CALL ERPNT1( ERSP8, 4, -1 ) RDDA 780
IF ( NSHPAN .GT. 30 ) CALL ERPNT1( ERSP9, 4, -1 ) RDDA 790
IF ( NBAR .GT. 130) CALL ERPNT1( ERSP3, 3, -1 ) RDDA 800
CALL ERPNT2 RDDA 810
REWIND 1 RDDA 820
1 READ(1,135) ICODE RDDA 830
135 FORMAT ( I4 ) RDDA 840
IF ( EOF, 1 ) 58, 4 RDDA 850
4 CONTINUE RDDA 860
GO TO ( 21, 22, 23, 24, 25, 26, 27), ICODE RDDA 870
21 CONTINUE RDDA 880
C READ CODE 1 DATA RDDA 890
READ(1,140) INDX,(RJXYZ(INDX,J),J=1,3),(IJRFM(INDX,J),J=1,6), RDDA 900
1 (IJALFM(INDX,J),J=1,2) RDDA 910
GO TO 1 RDDA 920
22 CONTINUE RDDA 930
C READ CODE 2 DATA RDDA 940
READ(1,140) INDX,(RJXYZ(INDX,J),J=1,3) RDDA 950
GO TO 1 RDDA 960
23 CONTINUE RDDA 970
C READ CODE 3 DATA RDDA 980
READ(1,140) INDX, RLIMTU(INDX), RLIMTL(INDX), DPRTIT(INDX) RDDA 990
GO TO 1 RDDA1000
24 CONTINUE RDDA1010
C READ CODE 4 DATA RDDA1020
READ(1,140) INDX, (FRCVCT(INDX,J),J=1,3) RDDA1030
GO TO 1 RDDA1040
25 CONTINUE RDDA1050
C READ CODE 5 DATA RDDA1060
READ(1,140) INDX, (RMTVCT(INDX,J),J=1,3) RDDA1070
GO TO 1 RDDA1080
26 CONTINUE RDDA1090
C READ CODE 6 DATA RDDA1100
READ(1,160) INDX, IBARP(INDX), IBARQ(INDX), IBARR(INDX), RDDA1110

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1          KS (INDX), BAREA(INDX), BARIN(INDX),      RDDA1120
2          BARIT(INDX), BARJ (INDX), BARYM(INDX),    RDDA1130
3          BARSM(INDX), RKN (INDX), RKT (INDX)      RDDA1140
GO TO 1                                           RDDA1150
C      READ CODE 7 DATA                          RDDA1160
27 CONTINUE                                       RDDA1170
READ(1,150) INDX, IAPAN(INDX), IBPAN(INDX), ICPAN(INDX), RDDA1180
1      IDPAN(INDX), VSPAN(INDX), ESPAN(INDX), TSPAN(INDX) RDDA1190
GO TO 1                                           RDDA1200
58 CONTINUE                                       RDDA1210
C      RDDA1220
C      CHECK BAR DEFINITION                       RDDA1230
C      RDDA1240
DO 60 I=1,NBAR                                    RDDA1250
IF (IBARP(I).GT.NJOINT.OR.IBARP(I).LT.1) CALL ERPNT1 (ERJN,5,-1) RDDA1260
IF (IBARQ(I).GT.NJOINT.OR.IBARQ(I).LT.1) CALL ERPNT1 (ERJN,5,-1) RDDA1270
IF (IBARP(I).EQ.IBARQ(I)) CALL ERPNT1 (ERPQ,4,-1)      RDDA1280
60 IF (IBARR(I).GT.NJPNT) CALL ERPNT1 (ERJR,7,-1)    RDDA1290
C      RDDA1300
C      CHECK JOINT DEFINITIONS FOR MISSING FORCE, MOMENT VECTORS, RDDA1310
C      AND RESTRAINT INFORMATION                 RDDA1320
DO 80 I=1,NJOINT                                    RDDA1330
DO 70 J=1,6                                          RDDA1340
K=IJRFM(I,J)                                        RDDA1350
IF (IABS(K).GT.NLIMIT) CALL ERPNT1 (ERLN,5,-1)      RDDA1360
IF (K.LE.0) GO TO 70                                RDDA1370
IF (RLIMTU(K).LE.RLIMTL(K)) CALL ERPNT1 (ERLT,5,-1) RDDA1380
70 CONTINUE                                         RDDA1390
IF (IJALFM(I,1).GT.NFORCE.OR.IJALFM(I,1).LT.0) CALL ERPNT1 (ERFR,6 RDDA1400
1,-1)                                               RDDA1410
IF (IJALFM(I,2).GT.NMOMNT.OR.IJALFM(I,2).LT.0) CALL ERPNT1 (ERMT,6 RDDA1420
1,-1)                                               RDDA1430
80 CONTINUE                                         RDDA1440
C      RDDA1450
C      CALL SHEAR PANEL SUBROUTINE              RDDA1460
C      RDDA1470
CALL SHRPAN                                        RDDA1480
IF ( NBAR .GT. 130) CALL ERPNT1( ERSP3, 3, -1 )    RDDA1490
CALL ERPNT2                                       RDDA1500
C      RDDA1510
C      INITIALIZE DATA                         RDDA1520
C      RDDA1530
NROW=6*NJOINT                                     RDDA1540
INDITR=NROW*3                                     RDDA1550
INDPLS=0                                          RDDA1560
INDWKT=0                                          RDDA1570
INDRKT=0                                          RDDA1580
MINRST=6                                          RDDA1590
IND1SL=0                                          RDDA1600
ERRTOL=.0001                                     RDDA1610
INDSFG=1                                          RDDA1620
INDSFL=1                                          RDDA1630
INDWTS = 0                                        RDDA1640
INDRLX = 2                                        RDDA1650
RELAXF = 1.                                       RDDA1660
ISFDIM = 12904                                    RDDA1670

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	DO 85 I = 1, NROW	RDDA1680
85	SOLVEC(I) = 0.0	RDDA1690
	INDNMA=0	RDDA1700
	IREDTO=NROW	RDDA1710
	INDWNM=0	RDDA1720
	NEIGVL = 5	RDDA1730
	TMAX=9999.	RDDA1740
C		RDDA1750
C	READ INDICATORS AND CONTROL DATA IN BY NAMELIST	RDDA1760
C		RDDA1770
	READ (5,INDATA)	RDDA1780
	WRITE(6,210) INDRKT,INDSFG,INDSFL,INDWKT,INDWTS,ISFDIM	RDDA1790
	IF (INDNMA.EQ.0) GO TO 100	RDDA1800
C	CHECK NORMAL MODE ANALYSIS DATA	RDDA1810
C	NEIGVL MUST BE A MULTIPLE OF 5	RDDA1820
	I = NEIGVL/5	RDDA1830
	IF (I *5 .NE. NEIGVL) NEIGVL = 5*(I+1)	RDDA1840
	WRITE(6,230) INDNMA,INDWNM,IREDTO,NEIGVL	RDDA1850
	WRITE(6,240) (AMASS(I),I=1,IREDTO)	RDDA1860
	NEIGVL = NEIGVL + 6	RDDA1870
	IF (NEIGVL .GT. IREDTO) CALL ERPNT1(EREGVL, 8, -1)	RDDA1880
	IF (IREDTO.GT.102) CALL ERPNT1 (ERORD,4,-1)	RDDA1890
	IF (IREDTO.EQ.NROW) GO TO 100	RDDA1900
	WRITE(6,250) (IRWKP(I),I=1,IREDTO)	RDDA1910
	IF (IREDTO.GT.NROW) CALL ERPNT1 (ERRED,4,-1)	RDDA1920
	J=IREDTO-1	RDDA1930
	DO 90 I=1,J	RDDA1940
90	IF (IRWKP(I).GE.IRWKP(I+1)) CALL ERPNT1 (ERRWKP,5,-1)	RDDA1950
100	CONTINUE	RDDA1960
	IF (INDRKT.EQ.0) GO TO 130	RDDA1970
C		RDDA1980
C	READ STRUCTURAL STIFFNESS MATRIX DATA AND LOCAL STIFFNESS	RDDA1990
C	AND TRANSFORMATION MATRICES	RDDA2000
	REWIND 9	RDDA2010
	READ (9) NROW1	RDDA2020
	IF (NROW.NE.NROW1) CALL ERPNT1 (ERFROW,5,1)	RDDA2030
	NROW1=NROW1+1	RDDA2040
	READ (9) (IROWL(I),I=1,NROW1)	RDDA2050
	JJ=1	RDDA2060
	DO 110 I=1,NROW	RDDA2070
	II=IROWL(I+1)-1	RDDA2080
	READ (9) (STRSTF(J),J=JJ,II)	RDDA2090
110	JJ=II+1	RDDA2100
	READ (9) STRSTF(JJ)	RDDA2110
	IF (INDNMA.NE.0) GO TO 130	RDDA2120
C		RDDA2130
C	FILE 2 IS NOT NEEDED IF OVERLAY 3 IS NOT CALLED	RDDA2140
C		RDDA2150
	REWIND 2	RDDA2160
	DO 120 I=1,NBAR	RDDA2170
	READ (9) (TRANSM(J,1),J=1,9)	RDDA2180
	WRITE (2) (TRANSM(J,1),J=1,9)	RDDA2190
	READ (9) (STIFML(J,1),J=1,144)	RDDA2200
120	WRITE (2) (STIFML(J,1),J=1,144)	RDDA2210
	IF (INDISL.NE.0) READ (9) (SOLVEC(I),I=1,NROW)	RDDA2220
130	IF (NJOINT.LT.2) CALL ERPNT1 (ERJNL,4,-1)	RDDA2230

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IF (NBAR.LT.1) CALL ERPNT1 (ERBN,4,-1)
IF (INDNMA.NE.0) GO TO 138
ERRTOL = ABS(ERRTOL)
WRITE(6,220) ERRTOL,IND1SL,INDITR,INDPLS,INDRLX,INDWKT,
1 MINRST,RELAXF,TMAX
IF (IND1SL.NE.0) WRITE(6,260) (SOLVEC(I),I=1,NROW)
138 CONTINUE
CALL ERPNT2
RETURN
C
140 FORMAT ( 1X14, 3E10.3,6I3,2I4 )
150 FORMAT ( 1X14, 4I3, 3E10.2 )
160 FORMAT (1X14, 3I3, A3, E8.3, 5E9.3, 2F5.2 )
210 FORMAT (33H1STRUCTURAL ANALYSIS CONTROL DATA //
* 13H0GENERAL DATA //
* 9H INDRKT =,I5,7X,39H, 1 IMPLIES READ ALL MATRICES FROM TAPE /
* 9H INDSFG =,I5,7X,37H, 0 IMPLIES WRITE GLOBAL BAR MATRICES /
* 9H INDSFL =,I5,7X,36H, 0 IMPLIES WRITE LOCAL BAR MATRICES /
* 9H INDWKT =,I5,7X,37H, 1 IMPLIES SAVE ALL MATRICES ON TAPE /
* 9H INDWTS =,I5,7X,40H, 1 IMPLIES PRINT TOTAL STIFFNESS MATRIX /
* 9H ISFDIM =,I7,5X,35H, MAX. STORAGE FOR STIFFNESS MATRIX /
* )
220 FORMAT (36H0DISPLACEMENT/ROTATION SOLUTION DATA //
* 9H ERRTOL =,E10.3,2X30H, ITERATION SOLUTION TOLERANCE /
* 9H IND1SL =,I5,7X,41H, 1 IMPLIES AN INITIAL SOLUTION IN SOLVEC /
* 9H INDITR =,I7,5X,32H, MAX. SOLUTION ITERATION CYCLES /
* 9H INDPLS =,I5,7X,31H, 1 IMPLIES CONSIDER PLASTICITY /
* 9H INDRXL =,I5,7X,28H, ITERATIVE SOLUTION METHOD /
* 9H INDWKT =,I5,7X,31H, 1 IMPLIES SAVE SOLVEC ON TAPE /
* 9H MINRST =,I5,7X,27H, MIN. ALLOWABLE RESTRAINTS /
* 9H RELAXF =,F7.2,5X20H, RELAXATION FACTOR /
* 9H TMAX =,F10.3, 2X31H, ITERATION CP TERMINATION TIME /
* )
230 FORMAT (20H0MODAL ANALYSIS DATA //
* 9H INDNMA =,I5,7X,26H, RUN NORMAL MODE ANALYSIS /
* 9H INDWNM =,I5,7X,35H, 1 IMPLIES WRITE MODE DATA ON TAPE /
* 9H IREDTO =,I5,7X,25H, ORDER OF REDUCED SYSTEM /
* 9H NEIGVL =,I5,7X,31H, REQUIRED NON-RIGID BODY MODES /
* )
240 FORMAT (56H AMASS IS THE DIAGONAL MASS VECTOR OF THE REDUCED SYSTR
*EM / (21X10E11.3))
250 FORMAT ( 54H IRWKP CONTAINS THE ROWS TO KEEP IN THE REDUCED SYSTEMR
* / (21X10I11))
260 FORMAT (30H SOLVEC IS THE SOLUTION VECTOR / (21X10E11.3) )
END
RDDA2240
RDDA2250
RDDA2260
RDDA2270
RDDA2280
RDDA2290
RDDA2300
RDDA2310
RDDA2320
RDDA2330
RDDA2340
RDDA2350
RDDA2360
RDDA2370
RDDA2380
RDDA2390
RDDA2400
RDDA2410
RDDA2420
RDDA2430
RDDA2440
RDDA2450
RDDA2460
RDDA2470
RDDA2480
RDDA2490
RDDA2500
RDDA2510
RDDA2520
RDDA2530
RDDA2540
RDDA2550
RDDA2560
RDDA2570
RDDA2580
RDDA2590
RDDA2600
RDDA2610
RDDA2620
RDDA2630
RDDA2640
RDDA2650
RDDA2660
RDDA2670
RDDA2680

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SUBROUTINE SHRPAN
COMMON COM(30)
EQUIVALENCE ( COM( 1), NJOINT )
EQUIVALENCE ( COM( 4), NBAR )
EQUIVALENCE ( COM( 29), NSHPAN )
COMMON / CMAIN /
1 RJXYZ (100,3), IJRFM ( 74,6), IJALFM( 74,2), FRCVCT( 74,3),
2 RMTVCT( 74,3), IBARP ( 130), IBARQ ( 130), IBARR ( 130),
3 KS ( 130), BAREA ( 130), BARIN ( 130), BARIT ( 130),
4 BARJ ( 130), BARYM ( 130), BARSM ( 130), RKN ( 130),
5 RKT ( 130), SOLVEC( 444), RLIMTU( 88), RLIMTL( 88),
6 DPRTIT( 88), IROWL ( 445), STRSTF(1)
COMMON/SP/IAPAN(30), IBPAN(30), ICPAN(30), IDPAN(30),
1 VSPAN(30), ESPAN(30), TSPAN(30)
DIMENSION VABC(12), AB(3), BC(3), CD(3), DA(3)
EQUIVALENCE(VABC(1),AB(1)),(VABC(4),BC(1)),(VABC(7),CD(1)),
1 (VABC(10),DA(1))
DIMENSION ERSP1(6), ERSP2(5)
DATA ERSP2 / 10H A SHEAR P, 10HANEL USES , 10HAN UNDEFIN,
1 10HED NODAL P, 10HOINT. /
DATA ERSP1 / 10H SHEAR PAN, 10HEL CORNER , 10HPOINTS DO ,
1 10HNOT DEFINE, 10H A RECTANG, 10HLE. /
DATA IBLK3S /10H /
IF ( NSHPAN .EQ. 0 ) RETURN
C CHECK SHEAR PANEL NODE POINT DEFINITIONS
DO 10 I = 1, NSHPAN
IF ( IAPAN(I) .GT. NJOINT ) CALL ERPNT1( ERSP2, 5, -1 )
IF ( IBPAN(I) .GT. NJOINT ) CALL ERPNT1( ERSP2, 5, -1 )
IF ( ICPAN(I) .GT. NJOINT ) CALL ERPNT1( ERSP2, 5, -1 )
IF ( IDPAN(I) .GT. NJOINT ) CALL ERPNT1( ERSP2, 5, -1 )
10 CONTINUE
CALL ERPNT2
DO 200 ISP = 1, NSHPAN
IA = IAPAN( ISP )
IB = IBPAN( ISP )
IC = ICPAN( ISP )
ID = IDPAN( ISP )
C CALCULATE THE VECTORS AB, BC, CD, AND DA TO CHECK
C CONDITIONS OF THE SHEAR PANEL
AL = 0.0
BL = 0.0
CL = 0.0
DO 20 I = 1, 3
AB(I) = RJXYZ( IB, I ) - RJXYZ( IA, I )
BC(I) = RJXYZ( IC, I ) - RJXYZ( IB, I )
CD(I) = RJXYZ( ID, I ) - RJXYZ( IC, I )
DA(I) = RJXYZ( IA, I ) - RJXYZ( ID, I )
AL = AL + AB(I)**2
BL = BL + BC(I)**2
CL = CL + CD(I)**2
20 CONTINUE
AL = SQRT( AL )
BL = SQRT( BL )
CL = SQRT( CL )
C

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C           IS THIS SHEAR PANEL A RECTANGLE                               SHRP 560
C
C           CHECK CORNER ANGLES (90)                                       SHRP 570
C           DO 40 I = 1, 7, 3                                             SHRP 580
C           CALL DOTX( ALL, VABC(I), VABC(I+3) )                          SHRP 590
C           IF ( ABS( ALL) .LT. 1.E-6 ) GO TO 40                          SHRP 600
C           CALL ERPNT1( ERSPI, 6, -1 )                                    SHRP 610
C           GO TO 200                                                       SHRP 620
40 CONTINUE                                                                SHRP 630
C           CHECK PLANE CONDITIONS                                         SHRP 640
C           CALL CROSSX(DA, AB, BC )                                       SHRP 650
C           CALL CROSSX(AB, BC, CD )                                       SHRP 660
C           CALL DOTX (BC(1), AB, AB )                                     SHRP 670
C           CALL DOTX (BC(2), DA, DA )                                     SHRP 680
C           BC(1)=SQRT( BC(1))                                             SHRP 690
C           BC(2) = SQRT( BC(2))                                           SHRP 700
C           DO 60 I = 1, 3                                                 SHRP 710
C           BC(3)=ABS(DA(I)/BC(2)-AB(I)/BC(1))                             SHRP 720
C           CD(1) = .5*BC(3) + 1.E-5                                       SHRP 730
C           IF ( BC(3)/ CD(1) .LE. .0001 ) GO TO 60                       SHRP 740
C           CALL ERPNT1 ( ERSPI, 6, -1 )                                    SHRP 750
C           GO TO 200                                                       SHRP 760
60 CONTINUE                                                                SHRP 770
C           IFRIST = 0                                                       SHRP 780
C           NBAR = NBAR + 1                                                 SHRP 790
C           IBARP(NBAR) = IA                                               SHRP 800
C           IBARQ(NBAR) = IC                                               SHRP 810
C           IBARR(NBAR) = -ID                                              SHRP 820
80 CONTINUE                                                                SHRP 830
C           AREA = (AL**2+BL**2)**1.5 * TSPAN(ISP)/(4.*AL*BL*(1.+VSPAN(ISP))) SHRP 840
C           KS (NBAR) = IBLK3S                                             SHRP 850
C           BAREA(NBAR) = AREA                                             SHRP 860
C           BARIN(NBAR) = 0.0                                             SHRP 870
C           BARIT(NBAR) = 0.0                                             SHRP 880
C           BARJ (NBAR) = 0.0                                             SHRP 890
C           BARSM(NBAR) = 0.0                                             SHRP 900
C           BARYM(NBAR) = ESPAN(ISP)                                       SHRP 910
C           IF ( IFRIST .NE. 0 ) GO TO 200                                  SHRP 920
C           IFRIST = 1                                                       SHRP 930
C           NBAR = NBAR + 1                                                 SHRP 940
C           IBARP(NBAR) = ID                                               SHRP 950
C           IBARQ(NBAR) = IB                                               SHRP 960
C           IBARR(NBAR) = -IA                                              SHRP 970
C           AL = CL                                                         SHRP 980
C           GO TO 80                                                         SHRP 990
200 CONTINUE                                                                SHRP1000
C           RETURN                                                           SHRP1010
C           END                                                               SHRP1020
C                                                                           SHRP1030

```

	SUBROUTINE CROSSX(S, A, B)	CROS 10
	DIMENSION A(1), B(1), S(1)	CROS 20
C	S .EQ. THE VECTOR OR CROSS PRODUCT OF VECTORS A AND B	CROS 30
	S(1) = A(2)*B(3) - A(3)*B(2)	CROS 40
	S(2) = -A(1)*B(3) + A(3)*B(1)	CROS 50
	S(3) = A(1)*B(2) - A(2)*B(1)	CROS 60
	RETURN	CROS 70
	ENTRY DOTX	CROS 80
C	S(1) .EQ. THE DOT OR INNER PRODUCT OF TWO VECTORS A AND B	CROS 90
	S(1) = A(1)*B(1) + A(2)*B(2) + A(3)*B(3)	CROS 100
	RETURN	CROS 110
	END	CROS 120

OVERLAY (SASLP, 2, 0)

OV20 10

```
PROGRAM STIFF
COMMON COM(30)
EQUIVALENCE ( COM ( 24), INDNMA )
EQUIVALENCE ( COM( 4 ), NBAR )
EQUIVALENCE ( COM ( 7 ), IBAR )
EQUIVALENCE ( COM( 9 ), INDSFL )
EQUIVALENCE ( COM( 10 ), INDSFG )
COMMON / CMAIN /
1  RJXYZ (100,3), IJRFM ( 74,6), IJALFM( 74,2), FRCVCT( 74,3),
2  RMTVCT( 74,3), IBARP ( 130), IBARQ ( 130), IBARR ( 130),
3  KS ( 130), BAREA ( 130), BARIN ( 130), BARIT ( 130),
4  BARJ ( 130), BARYM ( 130), BARSM ( 130), RKN ( 130),
5  RKT ( 130), SOLVEC( 444), RLIMTU( 88), RLIMTL( 88),
6  DPRIT( 88), IROWL ( 445), STRSTF(1)
DIMENSION ISTSTF(1)
EQUIVALENCE ( STRSTF(1), ISTSTF(1))
C
C      SET UP STRUCTURAL STIFFNESS MATRIX
C
CALL SETSTF
C
C      BUILD STRUCTURAL STIFFNESS MATRIX BY SUMMING BAR STIFFNESS
C                                     MATRICES
IF ( INDNMA .NE. 0 ) GO TO 6
CALL WRSTD1
6 CONTINUE
DO 30 IBAR=1,NBAR
C      COMPUTE STIFFNESS MATRIX AND TRANSFORMATION MATRIX FOR BAR
C      NUMBER IBAR
CALL STFTRN
C
C      SAVE LOCAL STIFFNESS M AND TRANSFORMATION M ON FILE TAPF2
C
IF ( INDNMA .NE. 0 ) GO TO 8
CALL WRSTDK
8 CONTINUE
IF ( INDSFL.NE.0) GO TO 10
CALL WRBDAT
10 CONTINUE
C      TRANSFORM BAR STIFFNESS MATRIX TO GLOBAL COORDINATE SYSTEM
CALL TRASMK
IF ( INDSFG.NE.0) GO TO 20
CALL BRSTRA
20 CONTINUE
C      PLACE TRANSFORMED BAR STIFFNESS MATRIX IN STRUCTURAL
C      STIFFNESS MATRIX
CALL STORMS
30 CONTINUE
RETURN
END
```

```

SUBROUTINE SETSTF
COMMON COM(30)
EQUIVALENCE ( COM( 4), NPAR )
EQUIVALENCE ( COM( 1 ), NJOINT )
EQUIVALENCE ( COM( 8), NROW )
EQUIVALENCE ( COM( 27), ISFDIM )
COMMON / CMAIN /
1  RJXYZ (100,3), IJRFM ( 74,6), IJALFM( 74,2), FRCVCT( 74,3),
2  RMTVCT( 74,3), IBARP ( 130), IBARQ ( 130), IBARR ( 130),
3  KS ( 130), BAREA ( 130), BARIN ( 130), BARIT ( 130),
4  BARJ ( 130), BARYM ( 130), BARSM ( 130), RKN ( 130),
5  RKT ( 130), SOLVEC( 444), RLIMTU( 88), RLIMTL( 88),
6  DPRTIT( 88), IROWL ( 445), STRSTF(1)
DIMENSION ILSET(74,74), ISTSTF(1)
EQUIVALENCE ( STRSTF(1), ISTSTF(1), ILSET(1,1) )
DIMENSION ILSETN(74)
DIMENSION NZPRW(74)
NULL=NJOINT/2
C
C      COMPILE AN ARRAY OF RELATED NODAL POINTS TO SET-UP STIFFNESS M
C
DO 5 I = 1, 74
5  NZPRW(I) = 0
DO 10 I=1,NULL
ILSET(I,1)=I
10 NZPRW(I) = 1
C
DO 20 I=1,NBAR
IP=IBARP(I)
IQ=IBARQ(I)
NRP=NZPRW(IP)+1
NRQ=NZPRW(IQ)+1
NZPRW(IP)=NRP
NZPRW(IQ)=NRQ
ILSET(IP,NRP)=IQ
20 ILSET(IQ,NRQ)=IP
C
NULL=NULL+1
DO 30 I=NULL,NJOINT
NRP=NZPRW(I)+1
NZPRW(I)=NRP
30 ILSET(I,NRP)=I
C
C      CALCULATE THE TOTAL NO. OF STORAGE WORDS NEEDED FOR THE
C      STIFFNESS MATRIX AND COMPARE WITH WHAT IS AVAILABLE
C
NULL = 0
DO 32 I = 1, NJOINT
32 NULL = NULL + NZPRW(I)
NULL = NULL * 42 + NJOINT*6 + 2
IF (NULL .LE. ISFDIM ) GO TO 34
WRITE(6, 200) NULL, ISFDIM
STOP
34 CONTINUE
C

```

```

SETS 10
SETS 20
SETS 30
SETS 40
SETS 50
SETS 60
SETS 70
SETS 80
SETS 90
SETS 100
SETS 110
SETS 120
SETS 130
SETS 140
SETS 150
SETS 160
SETS 170
SETS 180
SETS 190
SETS 200
SETS 210
SETS 220
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SETS 340
SETS 350
SETS 360
SETS 370
SETS 380
SETS 390
SETS 400
SETS 410
SETS 420
SETS 430
SETS 440
SETS 450
SETS 460
SETS 470
SETS 480
SETS 490
SETS 500
SETS 510
SETS 520
SETS 530
SETS 540
SETS 550

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C		SETS 560
C	SORT ABOVE ARRAY BY NODAL POINT	SETS 570
C		SETS 580
	DO 90 I=1,NJOINT	SETS 590
	NRQ=NZPRW(I)	SETS 600
	IF (NRQ.EQ.1) GO TO 90	SETS 610
40	NRP=0	SETS 620
	DO 80 J=2,NRQ	SETS 630
	IF (ILSET(I,J-1)-ILSET(I,J)) 80,50,70	SETS 640
50	NRQ=NRQ-1	SETS 650
	NZPRW(I)=NRQ	SETS 660
	DO 60 L=J,NRQ	SETS 670
60	ILSET(I,L)=ILSET(I,L+1)	SETS 680
	GO TO 40	SETS 690
70	NULL=ILSET(I,J)	SETS 700
	ILSET(I,J)=ILSET(I,J-1)	SETS 710
	ILSET(I,J-1)=NULL	SETS 720
	NRP=1	SETS 730
80	CONTINUE	SETS 740
	IF (NRP.NE.0) GO TO 40	SETS 750
90	CONTINUE	SETS 760
C		SETS 770
C	WRITE ARRAY ON FILE 1	SETS 780
C		SETS 790
	REWIND 1	SETS 800
	DO 100 I=1,NJOINT	SETS 810
	NRQ=NZPRW(I)	SETS 820
100	WRITE (1) (ILSET(I,J),J=1,NRQ)	SETS 830
C		SETS 840
C	BUILD STIFFNESS MATRIX STORAGE ARRAY	SETS 850
C		SETS 860
	REWIND 1	SETS 870
	NULL=1	SETS 880
	NROW=0	SETS 890
	DO 140 I=1,NJOINT	SETS 900
	NRP=NZPRW(I)	SETS 910
	NRQ=7*NRP+1	SETS 920
	READ (1) (ILSETN(J),J=1,NRP)	SETS 930
	NFIRST=NULL+1	SETS 940
	NROW=NROW+1	SETS 950
C		SETS 960
	L = NRQ + NULL - 1	SETS 970
	DO 110 J = NULL, L	SETS 980
110	STRSTF(J)=0.0	SETS 990
	ISTSTF(NULL)=-NROW	SETS1000
	IROWL(NROW)=NULL	SETS1010
	NULL=NFIRST	SETS1020
C		SETS1030
	DO 120 J=1,NRP	SETS1040
	ISTSTF(NULL)=(6*(ILSETN(J)-1)+1)	SETS1050
120	NULL=NULL+7	SETS1060
	NLAST=NULL-1	SETS1070
C		SETS1080
	DO 130 L=1,5	SETS1090
	NROW=NROW+1	SETS1100
	ISTSTF(NULL)=-NROW	SETS1110

	IROWL(NROW)=NULL	SETS1120
	NULL=NULL+1	SETS1130
	DO 130 J=NFIRST,NLAST	SETS1140
	STRSTF(NULL)=STRSTF(J)	SETS1150
130	NULL=NULL+1	SETS1160
140	CONTINUE	SETS1170
	ISTSTF(NULL)=-1	SETS1180
	IROWL(NROW+1)=NULL	SETS1190
	RETURN	SETS1200
200	FORMAT (16H0**** ERROR ****, I6, 22H WORDS OF STORAGE ARE ,	SETS1210
1	39HREQUIRED FOR THE STIFFNESS MATRIX (ONLY, I6 , /	SETS1220
2	22X 45H WORDS OF STORAGE ARE AVAILABLE WITH PRESENT ,	SETS1230
3	12HDIMENSIONING)	SETS1240
	END	SETS1250

```

SUBROUTINE STFTRN
COMMON COM(30)
EQUIVALENCE ( COM( 6), ITINJO )
EQUIVALENCE ( COM( 7), IBAR )
DIMENSION SIDEM(144)
EQUIVALENCE ( STIFML, SIDEM )
COMMON / CMAIN /
1  RJXYZ (100,3), IJRFM ( 74,6), IJALFM( 74,2), FRCVCT( 74,3),
2  RMTVCT( 74,3), IBARP ( 130), IBARQ ( 130), IBARR ( 130),
3  KS ( 130), BAREA ( 130), BARIN ( 130), BARIT ( 130),
4  BARJ ( 130), BARYM ( 130), BARSM ( 130), RKN ( 130),
5  RKT ( 130), SOLVEC( 444), RLIMTU( 88), RLIMTL( 88),
6  DPRIT( 88), IROWL ( 445), STRSTF(1)
DIMENSION RBARR(1)
EQUIVALENCE ( IBARR(1), RBARR(1) )
COMMON / OVER2 / STIFML(12,12), TRANSM(3,3)
DATA I3BLK / 3H /
ITINJO=1
IF (ABS(BARIT(IBAR))+ABS(BARIN(IBAR))+ABS(BARJ(IBAR))).EQ.0) ITINJO
1=0
IP=IBARP(IBAR)
IQ=IBARQ(IBAR)
IR =IABS( IBARR(IBAR))
C
C      COMPUTE UNIT VECTORS ALONG LOCAL AXIS FOR GLOBAL COORDINATE
C      TRANSFORMATION MATRIX
X=RJXYZ(IQ,1)-RJXYZ(IP,1)
Y=RJXYZ(IQ,2)-RJXYZ(IP,2)
Z=RJXYZ(IQ,3)-RJXYZ(IP,3)
BARLGT=SQRT(X*X+Y*Y+Z*Z)
IF ( IBARR(IBAR) .LT. 0 ) RBARR(IBAR) = -BARLGT
TRANSM(1,1)=X/BARLGT
TRANSM(1,2)=Y/BARLGT
TRANSM(1,3)=Z/BARLGT
IF (ITINJO.NE.0) GO TO 20
DO 10 I=2,3
DO 10 J=1,3
10  TRANSM(I,J)=0.0
GO TO 30
20  CONTINUE
X=RJXYZ(IR,1)-RJXYZ(IP,1)
Y=RJXYZ(IR,2)-RJXYZ(IP,2)
Z=RJXYZ(IR,3)-RJXYZ(IP,3)
DPD=TRANSM(1,1)*X+TRANSM(1,2)*Y+TRANSM(1,3)*Z
DDR=SQRT(X*X+Y*Y+Z*Z-DPD*DPD)
TRANSM(2,1)=(X-TRANSM(1,1)*DPD)/DDR
TRANSM(2,2)=(Y-TRANSM(1,2)*DPD)/DDR
TRANSM(2,3)=(Z-TRANSM(1,3)*DPD)/DDR
TRANSM(3,1)=TRANSM(1,2)*TRANSM(2,3)-TRANSM(2,2)*TRANSM(1,3)
TRANSM(3,2)=-TRANSM(1,1)*TRANSM(2,3)+TRANSM(2,1)*TRANSM(1,3)
TRANSM(3,3)=TRANSM(1,1)*TRANSM(2,2)-TRANSM(2,1)*TRANSM(1,2)
C
C      COMPUTE STIFFNESS MATRIX
C
30  CONTINUE

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C		STFT 560
C	ZERO RIGHT HALF OF STIFFNESS MATRIX	STFT 570
C		STFT 580
	DO 40 I=1,12	STFT 590
	DO 40 J=1,12	STFT 600
40	STIFML(I,J)=0.0	STFT 610
C		STFT 620
C	FILL LEFT HALF OF STIFFNESS MATRIX AND THE DIAGONAL	STFT 630
C		STFT 640
	S1DEM(1)=BARYM(IBAR)*PARFA(IBAR)/BARLGT	STFT 650
	S1DEM(37)=-S1DEM(1)	STFT 660
	S1DEM(40)=S1DEM(1)	STFT 670
	IF (ITINJO.EQ.0) GO TO 160	STFT 680
	S1DEM(14)=12.*BARYM(IBAR)*BARIN(IBAR)/BARLGT**3	STFT 690
	S1DEM(27)=12.*BARYM(IBAR)*BARIT(IBAR)/BARLGT**3	STFT 700
	S1DEM(50)=-S1DEM(14)	STFT 710
	S1DEM(53)=S1DEM(14)	STFT 720
	S1DEM(63)=-S1DEM(27)	STFT 730
	S1DEM(66)=S1DEM(27)	STFT 740
	S1DEM(79)=BARSM(IBAR)*PARJ(IBAR)/BARLGT	STFT 750
	S1DEM(87)=-6.*BARYM(IBAR)*BARIT(IBAR)/BARLGT**2	STFT 760
	S1DEM(90)=-S1DEM(87)	STFT 770
	S1DEM(92)=4.*BARYM(IBAR)*BARIT(IBAR)/BARLGT	STFT 780
	S1DEM(98)=6.*BARYM(IBAR)*BARIN(IBAR)/BARLGT**2	STFT 790
	S1DEM(101)=-S1DEM(98)	STFT 800
	S1DEM(105)=4.*BARYM(IBAR)*BARIN(IBAR)/BARLGT	STFT 810
	S1DEM(115)=-S1DEM(79)	STFT 820
	S1DEM(118)=S1DEM(79)	STFT 830
	S1DEM(123)=S1DEM(87)	STFT 840
	S1DEM(126)=-S1DEM(87)	STFT 850
	S1DEM(128)=2.*BARYM(IBAR)*BARIT(IBAR)/BARLGT	STFT 860
	S1DEM(131)=S1DEM(92)	STFT 870
	S1DEM(134)=S1DEM(98)	STFT 880
	S1DEM(137)=-S1DEM(98)	STFT 890
	S1DEM(141)=2.*BARYM(IBAR)*BARIN(IBAR)/BARLGT	STFT 900
	S1DEM(144)=S1DEM(105)	STFT 910
	IF (KS(IBAR).EQ.13BLK) GO TO 160	STFT 920
C	DECODE KS(IBAR)	STFT 930
C	IR = 1 IMPLIES SHEAR STRAIN IS INCLUDED	STFT 940
C	IP = 1 FREE 5P. IP = 2 FREE 5Q	STFT 950
C	IQ = 1 FREE 6P. IQ = 2 FREE 6Q	STFT 960
	DECODE (3,180,KS(IBAR))IR,IP,IQ	STFT 970
	X=BARSM(IBAR)*BAREA(IBAR)/BARLGT	STFT 980
	IF (RKN(IBAR).EQ.0.0) GO TO 50	STFT 990
	AN=STIFML(2,2)*RKN(IBAR)/X	STFT1000
	GO TO 60	STFT1010
50	AN=0.0	STFT1020
60	CONTINUE	STFT1030
	IF (RKT(IBAR).EQ.0.0) GO TO 70	STFT1040
	AT=STIFML(3,3)*RKT(IBAR)/X	STFT1050
	GO TO 80	STFT1060
70	AT=0.0	STFT1070
80	CONTINUE	STFT1080
	IF (IR.EQ.1) GO TO 120	STFT1090
	IF (IP.EQ.0) GO TO 120	STFT1100
C		STFT1110

C	Y	MOMENT FREE	STFT1120
C			STFT1130
		X=1./(4.+AT)	STFT1140
		STIFML(3,3)=X*STIFML(3,3)	STFT1150
		STIFML(3,6)=X*STIFML(3,6)	STFT1160
		STIFML(6,6)=X*STIFML(6,6)	STFT1170
		STIFML(8,11)=0.0	STFT1180
		IF (IP.EQ.2) GO TO 90	STFT1190
		I=11	STFT1200
		J=8	STFT1210
		GO TO 100	STFT1220
90		I=8	STFT1230
		J=11	STFT1240
100		CONTINUE	STFT1250
		Y=X+X	STFT1260
		STIFML(3,I)=Y*STIFML(3,I)	STFT1270
		STIFML(6,I)=Y*STIFML(6,I)	STFT1280
		Y=Y+X	STFT1290
		STIFML(I,I)=Y*STIFML(6,I)	STFT1300
		STIFML(3,J)=0.0	STFT1310
		STIFML(6,J)=0.0	STFT1320
		STIFML(J,J)=0.0	STFT1330
110		IF (IQ.NE.0) GO TO 130	STFT1340
C			STFT1350
C		Z MOMENT NOT FREE	STFT1360
C			STFT1370
		IF (RKN(IBAR).EQ.0.0) GO TO 160	STFT1380
		X=1./(1.+AN)	STFT1390
		STIFML(2,2)=X*STIFML(2,2)	STFT1400
		STIFML(2,5)=X*STIFML(2,5)	STFT1410
		STIFML(2,9)=X*STIFML(2,9)	STFT1420
		STIFML(2,12)=X*STIFML(2,12)	STFT1430
		STIFML(5,5)=X*STIFML(5,5)	STFT1440
		STIFML(5,9)=X*STIFML(5,9)	STFT1450
		STIFML(5,12)=X*STIFML(5,12)	STFT1460
		X=1.-3./(4.*(1.+1./AN))	STFT1470
		STIFML(9,9)=X*STIFML(9,9)	STFT1480
		STIFML(9,12)=(1.-3./(2.*(1.+1./AN)))*STIFML(9,12)	STFT1490
		STIFML(12,12)=X*STIFML(12,12)	STFT1500
		GO TO 160	STFT1510
C			STFT1520
C		Y MOMENT NOT FREE	STFT1530
C			STFT1540
120		IF (RKT(IBAR).EQ.0.0) GO TO 110	STFT1550
		X=1./(1.+AT)	STFT1560
		STIFML(3,3)=X*STIFML(3,3)	STFT1570
		STIFML(3,6)=X*STIFML(3,6)	STFT1580
		STIFML(3,8)=X*STIFML(3,8)	STFT1590
		STIFML(3,11)=X*STIFML(3,11)	STFT1600
		STIFML(6,6)=X*STIFML(6,6)	STFT1610
		STIFML(6,8)=X*STIFML(6,8)	STFT1620
		STIFML(6,11)=X*STIFML(6,11)	STFT1630
		X=1.-3./(4.*(1.+1./AT))	STFT1640
		STIFML(8,8)=X*STIFML(8,8)	STFT1650
		STIFML(8,11)=(1.-3./(2.*(1.+1./AT)))*STIFML(8,11)	STFT1660
		STIFML(11,11)=X*STIFML(11,11)	STFT1670

	GO TO 110	STFT1680
C	.	STFT1690
C	Z MOMENT FREE	STFT1700
C		STFT1710
130	X=1./(4.+AN)	STFT1720
	STIFML(2,2)=X*STIFML(2,2)	STFT1730
	STIFML(2,5)=X*STIFML(2,5)	STFT1740
	STIFML(5,5)=X*STIFML(5,5)	STFT1750
	STIFML(9,12)=0.0	STFT1760
	IF (IQ.EQ.2) GO TO 140	STFT1770
	I=12	STFT1780
	J=9	STFT1790
	GO TO 150	STFT1800
140	I=9	STFT1810
	J=12	STFT1820
150	CONTINUE	STFT1830
	Y=X+X	STFT1840
	STIFML(2,I)=Y*STIFML(2,I)	STFT1850
	STIFML(5,I)=Y*STIFML(5,I)	STFT1860
	Y=Y+X	STFT1870
	STIFML(I,I)=Y*STIFML(12,12)	STFT1880
	STIFML(2,J)=0.0	STFT1890
	STIFML(5,J)=0.0	STFT1900
	STIFML(J,J)=0.0	STFT1910
C		STFT1920
C	FILL LEFT HALF OF STIFFNESS MATRIX	STFT1930
C		STFT1940
160	CONTINUE	STFT1950
	DO 170 I=1,11	STFT1960
	II=I+1	STFT1970
	DO 170 J=II,12	STFT1980
170	STIFML(J,I)=STIFML(I,J)	STFT1990
	RETURN	STFT2000
C		STFT2010
180	FORMAT (311)	STFT2020
	END	STFT2030

	SUBROUTINE STORM	STOR 10
	COMMON COM(30)	STOR 20
	EQUIVALENCE (COM(6), ITINJO)	STOR 30
	EQUIVALENCE (COM(7), IBAR)	STOR 40
	COMMON / CMAIN /	STOR 50
1	RJXYZ (100,3), IJRFM (74,6), IJALFM(74,2), FRCVCT(74,3),	STOR 60
2	RMTVCT(74,3), IBARP (130), IBARQ (130), IBARR (130),	STOR 70
3	KS (130), BAREA (130), BARIN (130), BARIT (130),	STOR 80
4	BARJ (130), BARYM (130), BARSM (130), RKN (130),	STOR 90
5	RKT (130), SOLVEC(444), RLIMTU(88), RLIMTL(88),	STOR 100
6	DPRTIT(88), IROWL (445), STRSTF(1)	STOR 110
	DIMENSION I1STF(1)	STOR 120
	EQUIVALENCE (STRSTF(1), I1STF(1))	STOR 130
	COMMON / OVER2 / STIFML(12,12), TRANSM(3,3)	STOR 140
	DIMENSION IRC13(6), IRC24(6)	STOR 150
	DATA IRC13 / 1, 2, 3, 7, 8, 9 /	STOR 160
	DATA IRC24 / 4, 5, 6, 10, 11, 12 /	STOR 170
	ISP=(IBARP(IBAR)-1)*6+1	STOR 180
	ISQ=(IBARQ(IBAR)-1)*6+1	STOR 190
C		STOR 200
C	FIND INCREMENT FROM START OF ROW P TO COLUMNS P AND Q	STOR 210
C	CALL IPP AND IPQ	STOR 220
	J1=0	STOR 230
	II=IROWL(ISP)	STOR 240
	I1=II+1	STOR 250
10	IF (I1STF(I1).NE.ISP) GO TO 20	STOR 260
	IPP=I1-II	STOR 270
	J1=J1+1	STOR 280
	GO TO 30	STOR 290
20	IF (I1STF(I1).NE.ISQ) GO TO 30	STOR 300
	IPQ=I1-II	STOR 310
	J1=J1+1	STOR 320
30	I1=I1+7	STOR 330
	IF (J1.NE.2) GO TO 10	STOR 340
C		STOR 350
C	ADD TO THE 6 P ROWS	STOR 360
C		STOR 370
	IF (ITINJO.NE.0) GO TO 40	STOR 380
	I1=II+IPP+1	STOR 390
	STRSTF(I1)=STRSTF(I1)+STIFML(1,1)	STOR 400
	I1=II+IPQ+1	STOR 410
	STRSTF(I1)=STRSTF(I1)+STIFML(1,4)	STOR 420
	GO TO 70	STOR 430
40	CONTINUE	STOR 440
	DO 60 I=1,6	STOR 450
	IP=II+IPP	STOR 460
	IQ=II+IPQ	STOR 470
	I1=IRC13(I)	STOR 480
	DO 50 J=1,6	STOR 490
	J1=IRC13(J)	STOR 500
	JJ=IP+J	STOR 510
	STRSTF(JJ)=STRSTF(JJ)+STIFML(I1,J1)	STOR 520
	J1=IRC24(J)	STOR 530
	JJ=IQ+J	STOR 540
50	STRSTF(JJ)=STRSTF(JJ)+STIFML(I1,J1)	STOR 550

	J1=ISP+I	STOR 560
60	II=IROWL(J1)	STOR 570
C		STOR 580
C	FIND INCREMENT FROM START OF ROW Q TO COLUMNS P AND Q	STOR 590
C	CALL IPP	STOR 600
70	CONTINUE	STOR 610
	J1=0	STOR 620
	II=IROWL(ISQ)	STOR 630
	I1=II+1	STOR 640
80	IF (ISTSTF(I1).NE.ISQ) GO TO 90	STOR 650
	IQQ=I1-II	STOR 660
	J1=J1+1	STOR 670
	GO TO 100	STOR 680
90	IF (ISTSTF(I1).NE.ISP) GO TO 100	STOR 690
	IQP=I1-II	STOR 700
	J1=J1+1	STOR 710
100	I1=I1+7	STOR 720
	IF (J1.NE.2) GO TO 80	STOR 730
C		STOR 740
C	ADD TO THE 6 Q ROWS	STOR 750
C		STOR 760
	IF (ITINJO.NE.0) GO TO 110	STOR 770
	I1=II+IQQ+1	STOR 780
	STRSTF(I1)=STRSTF(I1)+STIFML(4,1)	STOR 790
	I1=II+IQP+1	STOR 800
	STRSTF(I1)=STRSTF(I1)+STIFML(4,4)	STOR 810
	RETURN	STOR 820
110	CONTINUE	STOR 830
	DO 130 I=1,6	STOR 840
	IQ=II+IQQ	STOR 850
	IP=II+IQP	STOR 860
	I1=IPC24(I)	STOR 870
	DO 120 J=1,6	STOR 880
	J1=IRC13(J)	STOR 890
	JJ=IP+J	STOR 900
	STRSTF(JJ)=STRSTF(JJ)+STIFML(I1,J1)	STOR 910
	J1=IRC24(J)	STOR 920
	JJ=IQ+J	STOR 930
120	STRSTF(JJ)=STRSTF(JJ)+STIFML(I1,J1)	STOR 940
	J1=ISQ+I	STOR 950
130	II=IROWL(J1)	STOR 960
	RETURN	STOR 970
	END	STOR 980

	SUBROUTINE TRASMK	TRAS	10
	COMMON COM(30)	TRAS	20
	EQUIVALENCE (COM(6), ITINJO)	TRAS	30
	COMMON / OVER2 / STIFML(12,12), TRANSM(3,3)	TRAS	40
	DIMENSION A(12), IJ(4), IJC(12)	TRAS	50
	DIMENSION IRC(12)	TRAS	60
	EQUIVALENCE(AXF, STIFML(1,1))	TRAS	70
	DATA IJ / 0,3,6,9 /	TRAS	80
	DATA IJC / 3*0, 3*3, 3*6, 3*9 /	TRAS	90
	DATA IRC / 1,2,3,1,2,3,1,2,3,1,2,3 /	TRAS	100
	ITINJO=1	TRAS	110
C		TRAS	120
C	IF THERE IS BENDING GO TO 8	TRAS	130
C		TRAS	140
	IF (ITINJO.NE.0) GO TO 20	TRAS	150
	DO 10 I=1,3	TRAS	160
	DO 10 J=1,3	TRAS	170
	AHOLD=AXF*TRANSM(1,J)*TRANSM(1,I)	TRAS	180
	STIFML(I,J)=AHOLD	TRAS	190
	STIFML(I,J+3)=-AHOLD	TRAS	200
	STIFML(I+3,J)=-AHOLD	TRAS	210
10	STIFML(I+3,J+3)=AHOLD	TRAS	220
	RETURN	TRAS	230
20	CONTINUE	TRAS	240
C		TRAS	250
C	MULTIPLY TRANSFORMATION MATRIX TIMES STIFFNESS MATRIX	TRAS	260
C	STORE RESULT IN STIFFNESS MATRIX	TRAS	270
C		TRAS	280
	DO 40 K=1,12	TRAS	290
	ICR=0	TRAS	300
	DO 30 IC=1,4	TRAS	310
	DO 30 I=1,3	TRAS	320
	ICR=ICR+1	TRAS	330
	A(ICR)=0.0	TRAS	340
	DO 30 J=1,3	TRAS	350
	JJ=J+IJ(IC)	TRAS	360
	IF (STIFML(K,JJ).EQ.0.) GO TO 30	TRAS	370
	A(ICR)=A(ICR)+STIFML(K,JJ)*TRANSM(J,I)	TRAS	380
30	CONTINUE	TRAS	390
	DO 40 J=1,12	TRAS	400
40	STIFML(K,J)=A(J)	TRAS	410
C		TRAS	420
C	.	TRAS	430
C	MULTIPLY THE RESULT OF THE ABOVE TIMES THE TRANSPOSE OF THE	TRAS	440
C	TRANSFORMATION MATRIX	TRAS	450
C		TRAS	460
	DO 60 K=1,12	TRAS	470
	DO 50 ICR=K,12	TRAS	480
	A(ICR)=0.0	TRAS	490
	I=IRC(ICR)	TRAS	500
	DO 50 J=1,3	TRAS	510
	JJ=J+IJC(ICR)	TRAS	520
50	A(ICR)=A(ICR)+STIFML(JJ,K)*TRANSM(J,I)	TRAS	530
	DO 60 J=K,12	TRAS	540
60	STIFML(J,K)=A(J)	TRAS	550
	DO 70 I=1,11	TRAS	560

```
IC=I+1
DO 70 J=IC,12
70 STIFML(I,J)=STIFML(J,I)
C
C     AS A RESULT OF THE ABOVE, STIFML CONTAINS THE STIFFNESS
C     MATRIX TRANSFORMED TO THE GLOBAL COORDINATE SYSTEM
C
RETURN
END
```

TRAS 570
TRAS 580
TRAS 590
TRAS 600
TRAS 610
TRAS 620
TRAS 630
TRAS 640
TRAS 650

```

SUBROUTINE WRBDAT
COMMON COM(30)
EQUIVALENCE ( COM( 7), IBAR )
COMMON / OVER2 / STIFML(12,12), TRANSM(3,3)
COMMON / CMAIN /
1 RJXYZ (100,3), IJRFM ( 74,6), IJALFM( 74,2), FRCVCT( 74,3),
2 RMTVCT( 74,3), IBARP ( 130), IBARQ ( 130), IBARR ( 130),
3 KS ( 130), BAREA ( 130), BARIN ( 130), BARIT ( 130),
4 BARJ ( 130), BARYM ( 130), BARSM ( 130), RKN ( 130),
5 RKT ( 130), SOLVEC( 444), RLIMTU( 88), RLIMTL( 88),
6 DPRIT( 88), IROWL ( 445), STRSTF(1)
C
C IF INDSFL .NE. 0, PRINT LOCAL STIFFNESS MATRIX AND
C TRANSFORMATION MATRIX FOR BAR(I)
C
WRITE (6,10) IBAR,IBARP(IBAR),IBARQ(IBAR),((STIFML(I,J),J=1,12),I=
11,12),((TRANSM(I,J),J=1,3),I=1,3)
RETURN
C
C IF INDSFG .NE. 0, PRINT THE TRANSFORMED LOCAL STIFFNESS MATRIX
C
ENTRY BRSTRA
WRITE (6,20) ((STIFML(I,J),J=1,12),I=1,12)
RETURN
C
10 FORMAT (13H1 BAR NUMBER I4/,25H0 POINT P IS NODAL POINT I4/,25H
10INT Q IS NODAL POINT I4/,18H0 STIFFNESS MATRIX//,12(12E11.3,/),23
2H0 TRANSFORMATION MATRIX//,3(3E11.3,/))
20 FORMAT (30H0 TRANSFORMED STIFFNESS MATRIX//,12(12E11.3,/))
END

```

	SUBROUTINE WRSTDK	WRSD 10
	COMMON / OVER2 / STIFML(12,12), TRANSM(3,3)	WRSD 20
C		WRSD 30
C	WRITE LOCAL STIFFNESS AND ASSOCIATED TRANSFORMATION	WRSD 40
C	MATRICES ON FILE 2 FOR LATER USE IN FMBARS (AND, IF	WRSD 50
C	INDWKT .NE. 0, IN PANDTK)	WRSD 60
C		WRSD 70
	WRITE (2) ((TRANSM(I,J),J=1,3),I=1,3)	WRSD 80
	WRITE (2) ((STIFML(I,J),J=1,12),I=1,12)	WRSD 90
	RETURN	WRSD 100
	ENTRY WRSTD1	WRSD 110
	REWIND 2	WRSD 120
	RETURN	WRSD 130
	END	WRSD 140

OVERLAY (SASLP, 3, 0)

OV30 10

```
PROGRAM FINIAL FINI 10
COMMON COM(30) FINI 20
EQUIVALENCE ( COM(16), INDWKT ) FINI 30
EQUIVALENCE ( COM( 8), NROW ) FINI 40
COMMON / CMAIN / FINI 50
1 RJXYZ (100,3), IJRFM ( 74,6), IJALFM( 74,2), FRCVCT( 74,3), FINI 60
2 RMTVCT(74,3), IBARP ( 130), IBARQ ( 130), IBARR ( 130), FINI 70
3 KS ( 130), BAREA ( 130), BARIN ( 130), BARIT ( 130), FINI 80
4 BARJ ( 130), BARYM ( 130), BARSM ( 130), RKN ( 130), FINI 90
5 RKT ( 130), SOLVEC( 444), RLIMTU( 88), RLIMTL( 88), FINI 100
6 DPRTIT( 88), IROWL ( 445), STRSTF(1) FINI 110
C FINI 120
C PANDTK, PRINTS THE NON ZERO ELEMENTS OF THE STRUCTURAL FINI 130
C STIFFNESS MATRIX AND CONDENSES THE BLOCK STORAGE WHERE FINI 140
C POSSIBLE FINI 150
CALL PANDTK FINI 160
C FINI 170
C SOLVE FINI 180
C FINI 190
CALL SOLVE FINI 200
IF (INDWKT.EQ.0) GO TO 10 FINI 210
WRITE (9) (SOLVEC(I),I=1,NROW) FINI 220
END FILE 9 FINI 230
10 CONTINUE FINI 240
C FINI 250
C CALCULATE LOCAL FORCE-MOMENT VECTORS FOR EACH BAR FINI 260
C FINI 270
CALL FMBARS FINI 280
RETURN FINI 290
END FINI 300
```

	SUBROUTINE PANDTK	PAND 10
	COMMON COM(30)	PAND 20
	EQUIVALENCE (COM(8), NROW)	PAND 30
	EQUIVALENCE (COM(16), INDWKT)	PAND 40
	EQUIVALENCE (COM(4), NBAR)	PAND 50
	EQUIVALENCE (COM (28), INDWTS)	PAND 60
	COMMON / CMAIN /	PAND 70
1	RJXYZ (100,3), IJRFM (74,6), IJALFM(74,2), FRCVCT(74,3),	PAND 80
2	RMTVCT(74,3), IBARP (130), IBARQ (130), IBARR (130),	PAND 90
3	KS (130), BAREA (130), BARIN (130), BARIT (130),	PAND 100
4	BARJ (130), BARYM (130), BARSM (130), RKN (130),	PAND 110
5	RKT (130), SOLVEC(444), RLIMTU(88), RLIMTL(88),	PAND 120
6	DPRTIT(88), IROWL (445), STRSTF(1)	PAND 130
	DIMENSION STF(519), ISTF(519)	PAND 140
	DIMENSION ISTSTF(1), STFDIA(444), STIFML(12,12), TRANSM(3,3)	PAND 150
	EQUIVALENCE (STRSTF(1), ISTSTF(1))	PAND 160
	EQUIVALENCE (BAREA (1), STFDIA(1))	PAND 170
	EQUIVALENCE (RJXYZ(1,1),STIFML(1,1))	PAND 180
	EQUIVALENCE (RJXYZ(50,2),TRANSM(1,1))	PAND 190
	COMMON / OVER3 / COM3(519)	PAND 200
	EQUIVALENCE (COM3(1), STF, ISTF)	PAND 210
	LDIA = 0	PAND 220
	LDIAS = 0	PAND 230
	IF (INDWKT.EQ.0) GO TO 10	PAND 240
C		PAND 250
C	WRITE THE NO. OF ROWS AND THEIR LOCATION IN STIFFNESS MATRIX	PAND 260
C	ON FILE 9	PAND 270
C		PAND 280
	REWIND 9	PAND 290
	WRITE (9) NROW	PAND 300
	J=NROW+1	PAND 310
	WRITE (9) (IROWL(I),I=1,J)	PAND 320
10	CONTINUE	PAND 330
C		PAND 340
C	PRINT NON ZERO ROW ELEMENTS AND DECIDE WHETHER THE ROW	PAND 350
C	SHOULD BE STORED IN BLOCK OR ELEMENT FORMAT	PAND 360
C		PAND 370
	LOCSTF=1	PAND 380
	DO 110 I=1,NROW	PAND 390
	ISUMNZ=0	PAND 400
	ISUMGP=0	PAND 410
20	LOCSTF=LOCSTF+1	PAND 420
	IF (ISTSTF(LOCSTF).LT.0) GO TO 50	PAND 430
	IC=ISTSTF(LOCSTF)-1	PAND 440
	ISUMGP=ISUMGP+1	PAND 450
	DO 40 J=1,6	PAND 460
	LOCSTF=LOCSTF+1	PAND 470
	IF (ABS(STRSTF(LOCSTF)).LT.1.E-9) GO TO 40	PAND 480
	ICC=IC+J	PAND 490
	IF (INDWTS .NE. 0) CALL WRSTRK (I, ICC, STRSTF(LOCSTF))	PAND 500
	IF (I.NE.ICC) GO TO 30	PAND 510
	LDIA=LOCSTF	PAND 520
	GO TO 40	PAND 530
30	ISUMNZ=ISUMNZ+1	PAND 540
40	CONTINUE	PAND 550

	GO TO 20	PAND 560
50	CONTINUE	PAND 570
C		PAND 580
C	IF INDWKT .NE. 0, WRITE STIFFNESS MATRIX IN BLOCK FORMAT	PAND 590
C	ON FILE 9 TO SAVE FOR FUTURE RUNS OF THE PROGRAM	PAND 600
C		PAND 610
C		PAND 620
	LCSTF1=IROWL(I)	PAND 630
	IF (INDWKT.EQ.0) GO TO 60	PAND 640
	NULL=LOCSTF-1	PAND 650
	WRITE (9) (STRSTF(J),J=LCSTF1,NULL)	PAND 660
60	CONTINUE	PAND 670
C		PAND 680
C	REMOVE THE DIAGONAL ELEMENTS OF THE STRUCTURAL STIFFNESS	PAND 690
C	MATRIX, AND SAVE THEM IN THE STFDIA ARRAY	PAND 700
C		PAND 710
	IF (LDIA .EQ. LDIAS) GO TO 70	PAND 720
	LDIAS = LDIA	PAND 730
	STFDIA(I)=STRSTF(LDIA)	PAND 740
	STRSTF(LDIA)=0.0	PAND 750
	GO TO 80	PAND 760
70	STFDIA(I)=0.0	PAND 770
80	CONTINUE	PAND 780
	SUMNZ = ISUMNZ	PAND 790
	SUMGP = ISUMGP	PAND 800
	IF (SUMNZ / SUMGP .GT. 3.) GO TO 110	PAND 810
C		PAND 820
C	IF HALF OR MORE OF THE ROW ELEMENTS ARE ZERO, REMOVE THEM	PAND 830
C	AND PLACE THAT ROW IN ELEMENT FORMAT	PAND 840
C		PAND 850
	NULL=1	PAND 860
	DO 90 J=1,ISUMGP	PAND 870
	LCSTF1=LCSTF1+1	PAND 880
	IC=ISTSTF(LCSTF1)-1	PAND 890
	DO 90 K=1,6	PAND 900
	LCSTF1=LCSTF1+1	PAND 910
	IF (ABS(STRSTF(LCSTF1)).LT.1.E-9) GO TO 90	PAND 920
	NULL=NULL+1	PAND 930
	ISTF(NULL)=IC+K	PAND 940
	NULL=NULL+1	PAND 950
	STF(NULL)=STRSTF(LCSTF1)	PAND 960
90	CONTINUE	PAND 970
	ISTF(1)=-NULL/2	PAND 980
	LCSTF1=IROWL(I)	PAND 990
	DO 100 J=1,NULL	PAND1000
	LCSTF1=LCSTF1+1	PAND1010
100	STRSTF(LCSTF1)=STF(J)	PAND1020
110	CONTINUE	PAND1030
C		PAND1040
C	PRINT LAST LINE IN WRSTRK	PAND1050
C		PAND1060
	IF (INDWTS .NE. 0) CALL WRSTRK(-1,-1,1.)	PAND1070
C		PAND1080
C	IF INDWKT .NE. 0, THEN THE LOCAL STIFFNESS AND TRANSFORMATION	PAND1090
C	MATRICES MUST BE SAVED ON FILE 9	PAND1100
C		PAND1110

```
IF (INDWKT.EQ.0) GO TO 130
REWIND 2
DO 120 I=1,NBAR
READ (2) (TRANSM(J,1),J=1,9)
WRITE (9) (TRANSM(J,1),J=1,9)
READ (2) (STIFML(J,1),J=1,144)
120 WRITE (9) (STIFML(J,1),J=1,144)
130 RETURN
END
```

PAND1120
PAND1130
PAND1140
PAND1150
PAND1160
PAND1170
PAND1180
PAND1190
PAND1200

```

SUBROUTINE SOLVE
COMMON COM(30)
EQUIVALENCE ( COM( 1), NJOINT )
EQUIVALENCE ( COM( 8), NROW )
EQUIVALENCE ( COM(11), IND1SL )
EQUIVALENCE ( COM(12), INDITR )
EQUIVALENCE ( COM(13), ERRTOL )
EQUIVALENCE ( COM(14), RELAXF )
EQUIVALENCE ( COM(15), INDR LX )
EQUIVALENCE ( COM(18), INDPLS )
EQUIVALENCE ( COM(19), MINRST )
EQUIVALENCE ( COM(26), TMAX )
COMMON / CMAIN /
1 RJXYZ (100,3), IJRFM ( 74,6), IJALFM( 74,2), FRCVCT( 74,3),
2 RMTVCT( 74,3), IBARP ( 130), IBARQ ( 130), IBARR ( 130),
3 KS ( 130), BAREA ( 130), BARIN ( 130), BARIT ( 130),
4 BARJ ( 130), BARYM ( 130), BARSM ( 130), RKN ( 130),
5 RKT ( 130), SOLVEC( 444), RLIMTU( 88), RLIMTL( 88),
6 DPRIT( 88), IROWL ( 445), STRSTF(1)
DIMENSION SOLVC2(444), ISTSTF(1), STFDIA(444)
DIMENSION RCG(444), APCG(444)
EQUIVALENCE ( RJXYZ(1,1), NONROW(1) )
EQUIVALENCE ( FRCVCT(1,1), SOLVC2(1) )
EQUIVALENCE ( STRSTF(1), ISTSTF(1))
EQUIVALENCE ( BAREA ( 1 ), STFDIA(1))
EQUIVALENCE ( BARJ(50), RCG(1) )
COMMON / OVER3 / FORCMG(1)
DIMENSION ALP(6), NONROW(300)
DIMENSION ERRFMN(6)
DIMENSION ERMNST(5)
DIMENSION ERDIA(5)
DIMENSION ERRL(6)
DATA ERRL /50H THE VARIABLE ASSOCIATED WITH THIS ROW HAS A CONST,
1 10HANT VALUE /
DATA ERDIA /10H THE DIAGO, 10HNAL ELEMEN, 10HT FOR THIS,
1 10H ROW CAN N, 10HOT BE ZERO /
DATA ERMNST /10H THE NUMBE, 10HR OF RESTR, 10HAINTS IS L,
1 10HESS THAN M, 10HINRST /
DATA ALP /10HX--FORCE , 10HY--FORCE , 10HZ--FORCE ,
1 10HX--MOMENT , 10HY--MOMENT , 10HZ--MOMENT /
DATA ERRFMN / 10H TOO MANY , 10HROWS ARE B, 10HEING REMOV,
1 10HED FOR THE, 10H SIZE OF N, 10HONROW /
WRITE (6,420)
C
C SET-UP MINIMUM ERROR FOR THE AITKEN DELTA SQUARED PROCESS
C
C IF (INDRLX.EQ.0) ERRTL2=AMIN1(ERRTOL*ERRTOL,1.E-10)
C
C CALCULATE FORCE AND MOMENT VECTOR
C
DO 10 I=1,300
10 NONROW(I)=0
DO 20 I=1,NROW
20 FORCMG(I)=0.0
NULL=0

```

```

SOLV 10
SOLV 20
SOLV 30
SOLV 40
SOLV 50
SOLV 60
SOLV 70
SOLV 80
SOLV 90
SOLV 100
SOLV 110
SOLV 120
SOLV 130
SOLV 140
SOLV 150
SOLV 160
SOLV 170
SOLV 180
SOLV 190
SOLV 200
SOLV 210
SOLV 220
SOLV 230
SOLV 240
SOLV 250
SOLV 260
SOLV 270
SOLV 280
SOLV 290
SOLV 300
SOLV 310
SOLV 320
SOLV 330
SOLV 340
SOLV 350
SOLV 360
SOLV 370
SOLV 380
SOLV 390
SOLV 400
SOLV 410
SOLV 420
SOLV 430
SOLV 440
SOLV 450
SOLV 460
SOLV 470
SOLV 480
SOLV 490
SOLV 500
SOLV 510
SOLV 520
SOLV 530
SOLV 540
SOLV 550

```

	DO 80 I=1,NJOINT	SOLV 560
	IC=IJALFM(I,1)	SOLV 570
	IF (IC.NE.0) GO TO 30	SOLV 580
	NULL=NULL+3	SOLV 590
	GO TO 50	SOLV 600
30	DO 40 J=1,3	SOLV 610
	NULL=NULL+1	SOLV 620
40	FORCMG(NULL)=FRCVCT(IC,J)	SOLV 630
50	IC=IJALFM(I,2)	SOLV 640
	IF (IC.NE.0) GO TO 60	SOLV 650
	NULL=NULL+3	SOLV 660
	GO TO 80	SOLV 670
60	DO 70 J=1,3	SOLV 680
	NULL=NULL+1	SOLV 690
70	FORCMG(NULL)=RMTVCT(IC,J)	SOLV 700
80	CONTINUE	SOLV 710
C		SOLV 720
C	STORE ROW NUMBERS NOT REQUIRED IN THE SOLUTION IN NONROW	SOLV 730
C	INDISL = 0 IMPLIES NO INITIAL SOLUTION IS AVAILABLE	SOLV 740
C		SOLV 750
	INDISL = 0	SOLV 760
	NULL=0	SOLV 770
	J=0	SOLV 780
	I=0	SOLV 790
	DO 130 K=1,NJOINT	SOLV 800
	DO 130 L=1,6	SOLV 810
	I=I+1	SOLV 820
	SOLVC2(I)=SOLVEC(I)	SOLV 830
	IF (IJRFM(K,L).EQ.0) GO TO 100	SOLV 840
	NULL=NULL+1	SOLV 850
	IF (NULL.GT.300) CALL ERPNT1 (ERRFMN,6,1)	SOLV 860
	NONROW(NULL)=I	SOLV 870
	IC=IA&S(IJRFM(K,L))	SOLV 880
	IF (DPRTIT(IC).EQ..0) GO TO 90	SOLV 890
	INDISL = 1	SOLV 900
	SOLVEC(I)=DPRTIT(IC)	SOLV 910
	SOLVC2(I)=DPRTIT(IC)	SOLV 920
	GO TO 130	SOLV 930
90	SOLVEC(I)=0.0	SOLV 940
	SOLVC2(I)=0.0	SOLV 950
	GO TO 130	SOLV 960
100	IF (STFDIA(I).EQ.0.0) GO TO 110	SOLV 970
	IF (INDRLX .EQ. 2) GO TO 130	SOLV 980
	IF (INDISL.NE.0) GO TO 130	SOLV 990
	GO TO 120	SOLV1000
110	CONTINUE	SOLV1010
	J=1	SOLV1020
	WRITE (6,430) I	SOLV1030
	GO TO 130	SOLV1040
120	CONTINUE	SOLV1050
	SOLVEC(I)=FORCMG(I)/STFDIA(I)	SOLV1060
	SOLVC2(I)=SOLVEC(I)	SOLV1070
130	CONTINUE	SOLV1080
	IF (J.NE.0) STOP	SOLV1090
	NNONRW=NULL	SOLV1100
C		SOLV1110

C		SOLV1120
C	ITERATION LOOP	SOLV1130
C		SOLV1140
140	IF (NONROW.LT.MINRST) CALL ERPNT1 (ERMST,5,1)	SOLV1150
	IF (INDRX .NE. 2) GO TO 1010	SOLV1160
	IF (INDISL .EQ. 1) INDISL = 1	SOLV1170
	FVMAX = 1.E-4	SOLV1180
	DO 1000 I = 1, NROW	SOLV1190
	IF (FVMAX .LT. ABS(FORCMG(I))) FVMAX = ABS(FORCMG(I))	SOLV1200
	APCG(I) = 0.0	SOLV1210
	RCG(I) = FORCMG(I)	SOLV1220
	IF (INDISL .NE. 0) GO TO 1000	SOLV1230
	SOLVC2(I) = 0.0	SOLV1240
	SOLVEC(I) = FORCMG(I)	SOLV1250
1000	CONTINUE	SOLV1260
1010	CONTINUE	SOLV1270
	ITRNO=0	SOLV1280
	IUPDAT=4	SOLV1290
	INDUPD=0	SOLV1300
	INDOPT=0	SOLV1310
	GO TO 150	SOLV1320
145	CONTINUE	SOLV1330
	DO 1015 I = 1, NROW	SOLV1340
	RCG(I) = FORCMG(I) - APCG(I)	SOLV1350
1015	SOLVEC(I) = RCG(I)	SOLV1360
150	ERRMAX=0.0	SOLV1370
	PAP =0.0	SOLV1380
	RCG2=0.0	SOLV1390
	NULL=1	SOLV1400
	ITRNO=ITRNO+1	SOLV1410
	DO 270 I=1,NROW	SOLV1420
	IF (I.NE.NONROW(NULL)) GO TO 160	SOLV1430
	NULL=NULL+1	SOLV1440
	GO TO 270	SOLV1450
160	SAVE=0.0	SOLV1460
	LOCSTF=IROWL(I)+1	SOLV1470
	IF (ISTSTF(LOCSTF).LT.0) GO TO 190	SOLV1480
C		SOLV1490
C	ROW IN BLOCK FORMAT, ONE COLUMN INDICATOR FOR 6 NON-ZERO	SOLV1500
C		ELEMENTS SOLV1510
170	ICOL=ISTSTF(LOCSTF)-1	SOLV1520
	DO 180 J=1,6	SOLV1530
	ICOL=ICOL+1	SOLV1540
	LOCSTF=LOCSTF+1	SOLV1550
180	SAVE=SAVE+STRSTF(LOCSTF)*SOLVEC(ICOL)	SOLV1560
	LOCSTF=LOCSTF+1	SOLV1570
	IF (ISTSTF(LOCSTF).LT.0) GO TO 210	SOLV1580
	GO TO 170	SOLV1590
C		SOLV1600
C	ROW IN ELEMENT FORMAT, ONE COLUMN INDICATOR FOR EACH NON-ZERO	SOLV1610
C		ELEMENT SOLV1620
190	NONELE=-ISTSTF(LOCSTF)	SOLV1630
	DO 200 J=1,NONELE	SOLV1640
	LOCSTF=LOCSTF+1	SOLV1650
	ICOL=ISTSTF(LOCSTF)	SOLV1660
	LOCSTF=LOCSTF+1	SOLV1670

200	SAVE=SAVE+STRSTF(LOCSTF)*SOLVEC(ICOL)	SOLV1680
210	CONTINUE	SOLV1690
	IF (INDRLX .NE. 2) GO TO 1100	SOLV1700
	APCG(I) = SAVE + STFDIA(I)*SOLVEC(I)	SOLV1710
	PAP = SOLVEC(I)*APCG(I) + PAP	SOLV1720
	RCG2 = RCG2 + RCG(I)*SOLVEC(I)	SOLV1730
	GO TO 270	SOLV1740
1100	CONTINUE	SOLV1750
C		SOLV1760
C	SUBTRACT SAVE FROM FORCMG(I) AND DIVIDE BY THE DIAGONAL	SOLV1770
C	ELEMENT (WHICH IS NOT CONSIDERED IN THE ABOVE CALCULATION)	SOLV1780
C	TO GET X(N+1)	SOLV1790
	IF (INDRLX.EQ.0) GO TO 220	SOLV1800
C		SOLV1810
C	USE OVERRELAXATION METHOD	SOLV1820
C		SOLV1830
	SAVE=RELAXF*(FORCMG(I)-SAVE)/STFDIA(I)+(1.-RELAXF)*SOLVEC(I)	SOLV1840
	ERR1=ABS(SOLVEC(I)-SAVE)	SOLV1850
	GO TO 250	SOLV1860
220	SAVE=(FORCMG(I)-SAVE)/STFDIA(I)	SOLV1870
	ERR1=ABS(SOLVEC(I)-SAVE)	SOLV1880
	IF (ITRNO.LT.IUPDAT) GO TO 250	SOLV1890
C		SOLV1900
C	USE THE AITKEN DELTA SQUARED PROCESS TO IMPROVE THE	SOLV1910
C	CONVERGENCE PROCESS	SOLV1920
	IF (ERR1/(ABS(SAVE)+1.E-14).LT.ERRTL2) GO TO 230	SOLV1930
	ERR2=ABS(SOLVC2(I)-SOLVEC(I))	SOLV1940
	IF (ERR1.GE.ERR2) GO TO 230	SOLV1950
	SAVE1=SAVE-(SAVE-SOLVEC(I))*2/(SAVE-2.*SOLVEC(I)+SOLVC2(I))	SOLV1960
	INDUPD=1	SOLV1970
	ERR1=ABS(SOLVFC(I)-SAVE1)	SOLV1980
	SOLVC2(I)=SAVE1	SOLV1990
	GO TO 240	SOLV2000
230	SOLVC2(I)=SAVE	SOLV2010
240	SOLVEC(I)=SAVE	SOLV2020
	ERR=ERR1/(ABS(SOLVC2(I))+1.E-12)	SOLV2030
	GO TO 260	SOLV2040
250	CONTINUE	SOLV2050
	SOLVC2(I)=SOLVEC(I)	SOLV2060
	SOLVEC(I)=SAVE	SOLV2070
	ERR=ERR1/(ABS(SAVE)+1.E-12)	SOLV2080
260	CONTINUE	SOLV2090
C		SOLV2100
	IF (ERR.GT.ERRMAX) ERRMAX=ERR	SOLV2110
270	CONTINUE	SOLV2120
	IF (INDRLX .NE. 2) GO TO 1270	SOLV2130
	IF (ITRNO .NE. 1) GO TO 1220	SOLV2140
	IF (IND1SL.NE. 0) GO TO 145	SOLV2150
1220	CONTINUE	SOLV2160
	IF (PAP .EQ. 0.) PAP = 1.E-9	SOLV2170
	ALPA = RCG2 / PAP	SOLV2180
	RCG2 = 0.0	SOLV2190
	DO 1240 J = 1, NROW	SOLV2200
	SOLVC2(J) = SOLVC2(J) + ALPA*SOLVEC(J)	SOLV2210
	RCG(J) = RCG(J) - ALPA*APCG(J)	SOLV2220
	ERRMAX = AMAX1(ABS(RCG(J)), ERRMAX)	SOLV2230

1240	RCG2 = RCG2 + RCG(J)*APCG(J)	SOLV2240
	ERRMAX = ERRMAX/ FVMAX	SOLV2250
	BETA = -RCG2 / PAP	SOLV2260
	DO 1260 J = 1, NROW	SOLV2270
1260	SOLVEC(J) = RCG(J) + BETA* SOLVEC(J)	SOLV2280
1270	CONTINUE	SOLV2290
	IF (INDUPD.EQ.0) GO TO 290	SOLV2300
	DO 280 K=1,NROW	SOLV2310
280	SOLVEC(K)=SOLVC2(K)	SOLV2320
	IUPDAT=IUPDAT+2	SOLV2330
	INDUPD=0	SOLV2340
290	CONTINUE	SOLV2350
C		SOLV2360
C	CHECK LOOP TERMINATION CONDITIONS	SOLV2370
C		SOLV2380
	IF (INDITR.EQ.ITRNO) GO TO 320	SOLV2390
	IF (INDITR.LT.ITRNO) GO TO 310	SOLV2400
	IF (INDOPT.GT.0) GO TO 320	SOLV2410
	CALL SECOND (TIME)	SOLV2420
	IF (TIME.GT.TMAX-5.) GO TO 300	SOLV2430
	IF (ERRMAX.GT.ERRTOL) GO TO 150	SOLV2440
	GO TO 320	SOLV2450
300	WRITE (6,440)	SOLV2460
	INDPLS=0	SOLV2470
	GO TO 320	SOLV2480
310	CONTINUE	SOLV2490
	WRITE (6,450) INDITR,ERRMAX	SOLV2500
320	CONTINUE	SOLV2510
	IF (INDRLX .NE. 2) GO TO 1310	SOLV2520
	DO 1300 J = 1, NROW	SOLV2530
1300	SOLVEC(J) = SOLVC2(J)	SOLV2540
	INDOPT = 2	SOLV2550
1310	CONTINUE	SOLV2560
	CALL PNTFMV (SOLVEC,NJOINT,1)	SOLV2570
	WRITE (6,460) ITRNO,ERRMAX	SOLV2580
	IUPDAT=IUPDAT+IUPDAT	SOLV2590
	INDOPT=INDOPT+1	SOLV2600
	IF (INDOPT.LT.2) GO TO 150	SOLV2610
C		SOLV2620
C	FIND THE COMPLETE FORCE-MOMENT VECTOR	SOLV2630
C		SOLV2640
	DO 370 K=1,NNONRW	SOLV2650
	SAVE=0.0	SOLV2660
	I=NONROW(K)	SOLV2670
	IF (SOLVEC(I).NE.0.0) SAVE=STFDIA(I)*SOLVEC(I)	SOLV2680
	LOCSTF=IROWL(I)+1	SOLV2690
	IF (ISTSTF(LOCSTF) .EQ. 0) GO TO 370	SOLV2700
	IF (ISTSTF(LOCSTF).LT.0) GO TO 350	SOLV2710
330	ICOL=ISTSTF(LOCSTF)-1	SOLV2720
	DO 340 J=1,6	SOLV2730
	ICOL=ICOL+1	SOLV2740
	LOCSTF=LOCSTF+1	SOLV2750
340	SAVE=SAVE+STRSTF(LOCSTF)*SOLVEC(ICOL)	SOLV2760
	LOCSTF=LOCSTF+1	SOLV2770
	IF (ISTSTF(LOCSTF).LT.0) GO TO 370	SOLV2780
	GO TO 330	SOLV2790

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350  NONELE=-ISTSTF(LOCSTF) SOLV2800
      DO 360 J=1,NONELE SOLV2810
      LOCSTF=LOCSTF+1 SOLV2820
      ICOL=ISTSTF(LOCSTF) SOLV2830
      LOCSTF=LOCSTF+1 SOLV2840
360  SAVE=SAVE+STRSTF(LOCSTF)*SOLVEC(ICOL) SOLV2850
370  FORCMG(I)=SAVE SOLV2860
      CALL PNTFMV (FORCMG,NJOINT,-1) SOLV2870
C SOLV2880
C     INDPLS = 0 IMPLIES NO PLASTICITY SOLV2890
C SOLV2900
      IF (INDPLS.EQ.0) RETURN SOLV2910
      NULL=0 SOLV2920
      INDBDL=NNONRW SOLV2930
      LOCSTF=0 SOLV2940
      DO 410 I=1,NJOINT SOLV2950
      DO 410 J=1,6 SOLV2960
      NULL=NULL+1 SOLV2970
      IF (IJRFM(I,J).EQ.0) GO TO 410 SOLV2980
      LOCSTF=LOCSTF+1 SOLV2990
      IF (IJRFM(I,J).LT.0) GO TO 410 SOLV3000
      ICOL=IJRFM(I,J) SOLV3010
      IF (RLIMTU(ICOL).GE.FORCMG(NULL)) GO TO 380 SOLV3020
      FORCMG(NULL)=RLIMTU(ICOL) SOLV3030
      GO TO 390 SOLV3040
380  IF (RLIMTL(ICOL).LE.FORCMG(NULL)) GO TO 410 SOLV3050
      FORCMG(NULL)=RLIMTL(ICOL) SOLV3060
390  NNONRW=NNONRW-1 SOLV3070
      IJRFM(I,J)=0 SOLV3080
      DO 400 K=LOCSTF,NNONRW SOLV3090
400  NONROW(K)=NONROW(K+1) SOLV3100
      LOCSTF=LOCSTF-1 SOLV3110
      IF (STFDIA(NULL).EQ.0.0) CALL ERPNT1 (ERDIA,5,-1) SOLV3120
      IF (SOLVEC(NULL).NE.0.0) CALL ERPNT1 (ERRL,6,-1) SOLV3130
      WRITE (6,470) ALP(J),I SOLV3140
410  CONTINUE SOLV3150
      CALL ERPNT2 SOLV3160
      IF (INDBDL.EQ.NNONRW) RETURN SOLV3170
      IND1SL = 1 SOLV3180
      GO TO 140 SOLV3190
C SOLV3200
420  FORMAT (1H1) SOLV3210
430  FORMAT (29H THE DIAGONAL ELEMENT FOR ROW,I4,16H CAN NOT BE ZERO) SOLV3220
440  FORMAT (66H SOLUTION ITERATIONS STOPPED BECAUSE JOB IS APPROACHING SOLV3230
1 TIME LIMIT) SOLV3240
450  FORMAT (10HOMORE THAN I5,28H ITERATIONS, MAXIMUM ERROR =E16.8) SOLV3250
460  FORMAT (1X I4,68H ITERATIONS WERE REQUIRED TO REACH A MAXIMUM RELAT SOLV3260
1IVE DIFFERENCE OF E14.7) SOLV3270
470  FORMAT (5H THE A10,37H RESTRAINT WAS VIOLATED FOR JOINT NO. I4) SOLV3280
      END SOLV3290

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	SUBROUTINE PNTFMV (ARRAY,NJOINT,IND)	PNTF 10
	COMMON COM(30)	PNTF 20
	EQUIVALENCE (COM(8), NROW)	PNTF 30
	DIMENSION ARRAY(1)	PNTF 40
	LCOUNT=70	PNTF 50
	I1=1	PNTF 60
	I2=6	PNTF 70
	DO 30 I=1,NJOINT	PNTF 80
	IF (LCOUNT.LT.56) GO TO 20	PNTF 90
C		PNTF 100
C	WRITE PAGE TITLE AT TOP OF NEXT PAGE	PNTF 110
C		PNTF 120
	LCOUNT=7	PNTF 130
	IF (IND.LT.0) GO TO 10	PNTF 140
	WRITE (6,50)	PNTF 150
	GO TO 20	PNTF 160
10	CONTINUE	PNTF 170
	WRITE (6,60)	PNTF 180
20	CONTINUE	PNTF 190
	WRITE (6,70) I,(ARRAY(K),K=I1,I2)	PNTF 200
	I1=I1+6	PNTF 210
	I2=I2+6	PNTF 220
	LCOUNT=LCOUNT+1	PNTF 230
30	CONTINUE	PNTF 240
	IF (IND.GT.0) RETURN	PNTF 250
	K=0	PNTF 260
	XF=0.0	PNTF 270
	YF=0.0	PNTF 280
	ZF=0.0	PNTF 290
	DO 40 I=1,NJOINT	PNTF 300
	K=K+1	PNTF 310
	XF=XF+ARRAY(K)	PNTF 320
	YF=YF+ARRAY(K+1)	PNTF 330
	ZF=ZF+ARRAY(K+2)	PNTF 340
40	K=K+5	PNTF 350
	WRITE (6,80) XF,YF,ZF	PNTF 360
	SAV = 0	PNTF 370
	DO 45 I = 1, NROW	PNTF 380
45	SAV = AMAX1(SAV, ABS(ARRAY(I)))	PNTF 390
	XF = XF/SAV *100.	PNTF 400
	YF = YF/SAV *100.	PNTF 410
	ZF = ZF/SAV *100.	PNTF 420
	WRITE(6,90) XF,YF,ZF	PNTF 430
	RETURN	PNTF 440
C		PNTF 450
50	FORMAT (1H1,36X39HNODAL POINT DISPLACEMENTS AND ROTATIONS/44X24HGLPNTF 460	
	10BAL COORDINATE SYSTEM/1H0,4X11HNODAL POINT,5X3(2X12HDISPLACMENT1PNTF 470	
	2X),3(4X8HROTATION3X)/7X7HNUMBER,2(14X1HX,14X1HY,14X1HZ)///) PNTF 480	
60	FORMAT (1H1,40X30HNODAL PCINT FORCES AND MOMENTS/44X24HGLOPAL COORPNTF 490	
	1DINATE SYSTEM/1H0,4X11HNODAL POINT,5X3(5X5HFORCE5X),3(5X6HMOMENT4XPNTF 500	
	2)/7X7HNUMBER 2(14X1HX,14X1HY,14X1HZ)///) PNTF 510	
70	FORMAT (8XI3,9X6(E15.6)) PNTF 520	
80	FORMAT (1H0,6X5HTOTAL,8X3F15.6) PNTF 530	
90	FORMAT (1H0 5X35HRELATIVE SOLUTION ERRORS IN PERCENT / PNTF 540	
	1 20X 3F15.5) PNTF 550	

END

PNTF 560

	SUBROUTINE FMBARS	FMBA 10
	COMMON COM(30)	FMBA 20
	EQUIVALENCE (COM(4), NBAR)	FMBA 30
	EQUIVALENCE (COM (29), NSHPAN)	FMBA 40
	COMMON / CMAIN /	FMBA 50
1	RJXYZ (100,3), IJRFM (74,6), IJALFM(74,2), FRCVCT(74,3),	FMBA 60
2	RMTVCT(74,3), IBARP (130), IBARQ (130), IBARR (130),	FMBA 70
3	KS (130), BAREA (130), BARIN (130), BARIT (130),	FMBA 80
4	BARJ (130), BARYM (130), BARSM (130), RKN (130),	FMBA 90
5	RKT (130), SOLVEC(444), RLIMTU(88), RLIMTL(88),	FMBA 100
6	DPRTIT(88), IROWL (445), STRSTF(1)	FMBA 110
	DIMENSION RBARR(1)	FMBA 120
	EQUIVALENCE (IBARR(1), RBARR(1))	FMBA 130
	DIMENSION STIFML(12,12), TRANSM(3,3)	FMBA 140
	EQUIVALENCE (RJXYZ(1,1), STIFML(1,1))	FMBA 150
	EQUIVALENCE (RJXYZ(50,2),TRANSM(1,1))	FMBA 160
	DIMENSION IPR(6), DISPL(12), DISPL2(12), FMLSTF(12)	FMBA 170
	DATA IPR / 1, 2, 3, 7, 8, 9 /	FMBA 180
	LCOUNT = 70	FMBA 190
	NPANEL = 0	FMBA 200
	LCUNT2 = 70	FMBA 210
	I1OR2 = 0	FMBA 220
C	CALCULATE THE NO. OF NON SHEAR PANEL BARS	FMBA 230
	NBARI = NBAR - 2*NSHPAN	FMBA 240
	REWIND 2	FMBA 250
	DO 50 I=1,NBAR	FMBA 260
	IF (I .LE. NBARI) GO TO 8	FMBA 270
	IF (LCUNT2 .LT. 56) GO TO 6	FMBA 280
	WRITE(6,80)	FMBA 290
	LCUNT2= 7	FMBA 300
6	CONTINUE	FMBA 310
	BARLGT = ABS(RBARR(I))	FMBA 320
	I1OR2 = I1OR2 + 1	FMBA 330
	GO TO 12	FMBA 340
8	CONTINUE	FMBA 350
	IF (LCOUNT.LT.56) GO TO 10	FMBA 360
C		FMBA 370
C	WRITE PAGE TITLE AT TOP OF NEXT PAGE	FMBA 380
C		FMBA 390
	LCOUNT=7	FMBA 400
	WRITE (6,60)	FMBA 410
10	CONTINUE	FMBA 420
	IR=IBARR(I)	FMBA 430
12	CONTINUE	FMBA 440
	IP=IBARP(I)	FMBA 450
	IQ=IBARQ(I)	FMBA 460
	READ (2) ((TRANSM(K,J),J=1,3),K=1,3)	FMBA 470
	READ (2) ((STIFML(K,J),J=1,12),K=1,12)	FMBA 480
C		FMBA 490
C	SET UP DISPLACEMENT VECTOR	FMBA 500
C		FMBA 510
	JJ=(IQ-1)*6	FMBA 520
	II=(IP-1)*6	FMBA 530
	DO 20 K=1,6	FMBA 540
	II=II+1	FMBA 550

	I1=IPR(K)	FMBA 560
	DISPL(I1)=SOLVEC(I1)	FMBA 570
	JJ=JJ+1	FMBA 580
	I1=I1+3	FMBA 590
20	DISPL(I1)=SOLVEC(JJ)	FMBA 600
C		FMBA 610
C	TRANSFORM DISPLACEMENT VECTOR	FMBA 620
C		FMBA 630
	II=-3	FMBA 640
	IROW=0	FMBA 650
	DO 30 K=1,4	FMBA 660
	II=II+3	FMBA 670
	DO 30 J=1,3	FMBA 680
	IROW=IROW+1	FMBA 690
	DISPL2(IROW)=0.0	FMBA 700
	DO 30 JJ=1,3	FMBA 710
	I1=II+JJ	FMBA 720
30	DISPL2(IROW)=DISPL2(IROW)+TRANSM(J,JJ)*DISPL(I1)	FMBA 730
C		FMBA 740
C	CALCULATE LOCAL FORCE-MOMENT VECTOR	FMBA 750
C		FMBA 760
	DO 40 K=1,12	FMBA 770
	FMLSTF(K)=.0	FMBA 780
	DO 40 J=1,12	FMBA 790
40	FMLSTF(K)=FMLSTF(K)+STIFML(K,J)*DISPL2(J)	FMBA 800
	IF (I .LE. NBARI) GO TO 48	FMBA 810
	SHFLOW = 2.*FMLSTF(1) / BARLGT	FMBA 820
	IF (I1OR2 .NE. 1) GO TO 42	FMBA 830
	IPS = IP	FMBA 840
	IQS = IQ	FMBA 850
	SHFSV = SHFLOW	FMBA 860
	GO TO 50	FMBA 870
42	CONTINUE	FMBA 880
	NPANEL = NPANEL + 1	FMBA 890
	SHFSV = (ABS(SHFLOW) + ABS(SHFSV)) * .5	FMBA 900
	SHFSV = SIGN(SHFSV, SHFLOW)	FMBA 910
	WRITE(6,90) NPANEL, IPS, IQ, IQS, IP, SHFSV	FMBA 920
	I1CR2 = 0	FMBA 930
	LCUNT2 = LCUNT2 + 2	FMBA 940
	GO TO 50	FMBA 950
48	CONTINUE	FMBA 960
	WRITE (6,70) I,IP,IQ,IR,(FMLSTF(K),K=1,3),(FMLSTF(K),K=7,9),(FMLSTF(K),K=4,6),(FMLSTF(K),K=10,12)	FMBA 970
	LCOUNT=LCOUNT+3	FMBA 980
	CONTINUE	FMBA 990
50	CONTINUE	FMBA1000
	RETURN	FMBA1010
C		FMBA1020
60	FORMAT (1H1,49X22H PAR FORCES AND MOMENTS/49X24H LOCAL COORDINATE SYSTEMS/1H0,29H BAR NODAL POINT NUMBERS,11X3(5X5H FORCE 5X),3(5X6H MOMENT 4X)/30H NUMBER P Q R ,4X2(14X1HX,14X1HY,14X1HZ)///)	FMBA1030
		FMBA1040
		FMBA1050
		FMBA1060
70	FORMAT (3XI3,6XI3,4XI3,4XI3,11H POINT P ,6E15.6/29X,11H POINT Q ,6E15.6/)	FMBA1070
		FMBA1080
80	FORMAT(1H15I17HSHEAR PANEL FLOWS /	FMBA1090
	1 49X24H LOCAL COORDINATE SYSTEMS /	FMBA1100
	246H0 PANEL NODAL POINT NUMBERS EQUIVALENT /	FMBA1110

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346H NUMBER  A   B   C   D   SHEAR FLOW,  /// )
90 FORMAT ( 1H0, I5, I8, 3I5, 7X E10.3 , 7XE10.3 )
END
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FMBA1120
FMBA1130
FMBA1140
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PROGRAM NLMDAL	NLMD 10
DIMENSION SM(444)	NLMD 20
DIMENSION AMASSV(102)	NLMD 30
DIMENSION RJX(1), RJY(1), RJZ(1)	NLMD 40
DIMENSION AMASS(102), IRWKP(103)	NLMD 50
DIMENSION EGVALU(444), EGVECT(2220), WORK1(444),	NLMD 60
1 WORK2(444), ADIA(444), BOFDIA(444)	NLMD 70
DIMENSION ROWNXT(444), IRWTOR(444), T(444)	NLMD 80
COMMON COM(30)	NLMD 90
EQUIVALENCE (COM(8), NROW)	NLMD 100
EQUIVALENCE (COM(21), NRWTOR)	NLMD 110
EQUIVALENCE (COM(22), IREDTO , MORDER)	NLMD 120
EQUIVALENCE (COM(23), NEIGVL)	NLMD 130
COMMON / CMAIN /	NLMD 140
1 RJXYZ (100,3), IJRFM (74,6), IJALFM(74,2), FRCVCT(74,3),	NLMD 150
2 RMTVCT(74,3), IBARP (130), IBARQ (130), IBARR (130),	NLMD 160
3 KS (130), BAREA (130), BARIN (130), BARIT (130),	NLMD 170
4 BARJ (130), BARYM (130), BARSM (130), RKN (130),	NLMD 180
5 RKT (130), SOLVEC(444), RLIMTU(88), RLIMTL(88),	NLMD 190
6 DPRTIT(88), IROWL (445), STRSTF(1)	NLMD 200
EQUIVALENCE (RJXYZ(1,1),RJX (1)),(RJXYZ(1,2),RJY(1))	NLMD 210
EQUIVALENCE (RJXYZ(1,3),RJZ(1)),(FRCVCT(1,1),SM(1), ADIA(1))	NLMD 220
EQUIVALENCE (BAREA(1),ROWNXT(1)), (SOLVEC(1), IRWTOR(1))	NLMD 230
EQUIVALENCE (RLIMTU(1),AMASS(1)), (RLIMTL(15), IRWKP(1))	NLMD 240
EQUIVALENCE (STRSTF(1), ISTSTF)	NLMD 250
EQUIVALENCE (STRSTF(5400), EGVALU)	NLMD 260
EQUIVALENCE (STRSTF(5500), EGVECT)	NLMD 270
EQUIVALENCE (STRSTF(7964), WORK1)	NLMD 280
EQUIVALENCE (STRSTF(8408), WORK2)	NLMD 290
EQUIVALENCE (STRSTF(9740), T)	NLMD 300
C INITIALIZE THE PRINT ROUTINE	NLMD 310
CALL PRNTO	NLMD 320
DO 10 I=1,MORDER	NLMD 330
10 AMASSV(I)=AMASS(I)	NLMD 340
ORDER=MORDER	NLMD 342
SUM=0.0	NLMD 344
DO 11 I=1,MORDER	NLMD 346
11 SUM=SUM+AMASS(I)	NLMD 348
SMMAX=SUM/ORDER	NLMD 350
DO 12 I=1,MORDER	NLMD 352
12 AMASS(I)=AMASS(I)/SMMAX	NLMD 354
C REDUCE STIFFNESS MATRIX	NLMD 356
CALL REDUCE	NLMD 358
KOUNT=MORDER*MORDER	NLMD 360
SUM=0.0	NLMD 362
DO 13 I=1,MORDER	NLMD 364
13 IDIAG=(I-1)*MORDER+I	NLMD 366
SUM=SUM+STRSTF(IDIAG)	NLMD 368
SKMAX=SUM/ORDER	NLMD 370
DO 14 I=1,KOUNT	NLMD 375
14 STRSTF(I)=STRSTF(I)/SKMAX	NLMD 380
C T	NLMD 390

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C      MATRIX = ((M)**-.5) *(K)*((M)**-.5) NLMD 400
C CALL STFMAS (STRSTF,AMASS,MORDER) NLMD 410
C NLMD 420
C      HOUSEHOLDER METHOD TO OBTAIN MATRIX IN TRIPLE-DIAGONAL FORM NLMD 430
C NLMD 440
C CALL TRIDIA (MORDER,STRSTF,ADIA,BOFDIA) NLMD 450
C NLMD 460
C      FIND THE NEIGVL + 6 SMALLEST EIGENVALUES NLMD 470
C NLMD 480
C CALL EIGVAL (MORDER,EGVALU,ADIA,BOFDIA,WORK1,WORK2,NEIGVL) NLMD 490
C NLMD 500
C      FIND THE EIGENVECTORS FOR THE NEIGVL EIGENVALUES ABOVE NLMD 510
C      FOR MATRIX NLMD 520
C      RATIO=SKMAX/SMAX NLMD 530
C      IGV1 = 2 NLMD 540
C      IGV2 = 6 NLMD 550
C      REWIND 9 NLMD 560
15 CONTINUE NLMD 570
C      IGV1 = IGV1 + 5 NLMD 580
C      IGV2 = IGV2 + 5 NLMD 590
C      J=1 NLMD 600
C      DO 20 I=IGV1, IGV2 NLMD 610
C      CALL EIGVEC (MORDER,EGVALU(I),STRSTF,ADIA,BOFDIA,EGVECT(J),WORK1,WORK2) NLMD 620
21 EGVALU(I)=RATIO*EGVALU(I) NLMD 630
20 J=J+NROW NLMD 640
C NLMD 650
C      CALCULATE EIGENVECTOR FOR REDUCED MATRIX FORM NLMD 660
C NLMD 670
C      K=0 NLMD 680
C      DO 50 M=1,5 NLMD 690
C      SAVE= 0.0 NLMD 700
C      DO 30 J=1,MORDER NLMD 710
C      I=K+J NLMD 720
C      EGVECT(I)=AMASS(J)*EGVECT(I) NLMD 730
C      IF (ABS(SAVE).LT.ABS(EGVECT(I))) SAVE=EGVECT(I) NLMD 740
30 CONTINUE NLMD 750
C      DO 40 J=1,MORDER NLMD 760
C      I=K+J NLMD 770
40 EGVECT(I)=EGVECT(I)/SAVE NLMD 780
50 K=K+NROW NLMD 790
C      CALL PRNT1 (EGVALU,EGVECT,AMASSV,IRWKP,NROW,MORDER,IRWTOR,WORK1) NLMD 800
C NLMD 810
C      TAKE ABOVE EIGENVECTORS AND USE THE TRANSFORMATION MATRICES NLMD 820
C      TO GET FINAL EIGENVECTORS NLMD 830
C      IF (IRFDTO.EQ.NROW) GO TO 60 NLMD 840
C      CALL FNALEV (EGVECT,AMASS,MORDER,NEIGVL,NROW,NRWTOR,T,WORK1,IRWKP, NLMD 850
1 IRWTOR) NLMD 860
60 CONTINUE NLMD 870
C NLMD 880
C      PRINT NLMD 890
C NLMD 900
C      CALL PRNT2 (EGVALU,EGVECT,AMASSV,IRWKP,NROW,MORDER,IRWTOR,WORK1) NLMD 910
C      IF ( IGV2 .LT. NEIGVL ) GO TO 15 NLMD 920
C      RETURN NLMD 930
C      END NLMD 940

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	IDENT	PUNPACK			PACK 010	
	ENTRY	PACK,UNPACK			PACK 020	
	VFD	42/4CPACK,18/3			PACK 030	
PACK	DATA	0			PACK 040	
	SA1	B2	***		PACK 050	
	SB2	X1	*	B2 = NL	PACK 060	
	SB4	B0		B4 = 0	INITIALLY	PACK 070
	SB5	9		B5 = 9	INITIALLY	PACK 080
	SB6	60		B6 = 60	INITIALLY	PACK 090
	SB7	1		B7 = 1	INITIALLY	PACK 100
	SA5	BITOONE		X5 = -0		PACK 110
WORDITS	IX7	X7-X7		X7 = 0		PACK 120
	SA2	BIT1ONE		X2 = 02000000000000000000		PACK 130
	SA4	B1+B4		X4 = INPUTA(1 + B4)		PACK 140
	ZR	X4,FSTBIT		IF (X4 .EQ. 0.0) GO TO FSTBIT		PACK 150
	BX7	X5		X7 = X5		PACK 160
	BX6	X4		X6 = X4		PACK 170
	SA6	B3+B5		OUTPUTA(1 + B5) = X6		PACK 180
	SB5	B5+1		B5 = B5 + 1		PACK 190
	ZR	FSTBIT		GO TO FSTBIT		PACK 200
LOOP	SA4	B1+B4		X4 = INPUTA(B1 + B4)		PACK 210
	ZR	X4,EQZERO		IF (X4 .EQ. 0.0) GO TO EQZERO		PACK 220
	BX7	X7+X2		X7 = LOGICAL SUM X7 AND X2		PACK 230
	BX6	X4		X6 = X4		PACK 240
	SA6	B3+B5		OUTPUTA(1 + B5) = X6		PACK 250
	SB5	B5+1		B5 = B5+1		PACK 260
EQZERO	AX2	1		CIRCULAR SHIFT X2 1 BIT RIGHT		PACK 270
FSTBIT	SB4	B4+1		B4 = B4 + 1		PACK 280
	EQ	B4,B2,ENDCHK		IF (B4 .EQ. B2) GO TO ENDCHK		PACK 290
	NE	B4,B6,LOOP		IF (B4 .NE. B6) GO TO LOOP		PACK 300
	SA7	B3+B7		OUTPUTA(1 + B7) = X7		PACK 310
	SB7	B7+1		B7 = B7 + 1		PACK 320
	SB6	B6+60		B6 = B6 + 60		PACK 330
	ZR	WORDITS		GO TO WORDI		PACK 340
ENDCHK	SA7	B3+B7		OUTPUTA(1 + B7) = X7		PACK 350
	SX6	B5		X6 = B5		PACK 360
	SA6	B3		OUTPUTA(1) = X6		PACK 370
	ZR	PACK		RETURN		PACK 380
	VFD	42/6CUNPACK,18/3				PACK 390
UNPACK	DATA	0				PACK 400
	SA2	B2	***	B2 = NF		PACK 410
	SB2	X2	*			PACK 420
	SB4	9		B4 = 9	INITIALLY	PACK 430
	SB5	0		B5 = 0	INITIALLY	PACK 440
	SA1	B1	***			PACK 450
	SB6	X1	*	B6 = LENGTH OF INPUTA ARRAY		PACK 460
	SB7	0		B7 = 0	INITIALLY	PACK 470
	SX5	1		X5 = 1	INITIALLY	PACK 480
	IX6	X1-X1		X6 = 0	INITIALLY	PACK 490
WORDJTS	EO	B4,B6,LOOPJ2		IF (B4 .EQ. B6) GO TO LOOPJ2		PACK 500
	SA4	X5+B1		X4 = INPUTA(1 + X5)		PACK 510
	SX5	X5+1		X5 = X5 + 1		PACK 520
	SB7	B7+60		B7 = B7 + 60		PACK 530
LOOPJ1	PL	X4,X4PLUS		IF (X4 .POSITIVE.) GO TO X4PLUS		PACK 540
	SA3	B1+B4		X3 = INPUTA(1 + B4)		PACK 550

	BX7	X3	X7 = X3	PACK 560
	SA7	B3+B5	OUTPUTA(1 + B5) = X7	PACK 570
	SB4	B4+1	B4 = B4+1	PACK 580
	ZR	COMCOD1	GO TO COMCOD1	PACK 590
X4PLUS	SA6	B3+B5	OUTPUTA(1 + B5) = X6 = 0.0	PACK 600
COMCOD1	LX4	1	CIRCULAR SHIFT X4 1 BIT LEFT	PACK 610
	SB5	B5+1	B5 = B5+1	PACK 620
	EQ	B5,B2,UNPACK	IF (B5 .EQ. 32) RETURN	PACK 630
	EQ	B5,B7,WORDJTS	IF (B5 .EQ. B7) GO TO WORDJTS	PACK 640
	NE	B4,B6,LOOPJ1	IF (B4 .NE. B6) GO TO LOOPJ1	PACK 650
LOOPJ2	SA6	B3+B5	OUTPUTA(1 + B5) = X6 = 0.0	PACK 660
	SB5	B5+1	B5 = B5 + 1	PACK 670
	NE	B5,B2,LOOPJ2	IF (B5 .NE. 32) GO TO LOOPJ2	PACK 680
	ZR	UNPACK	RETURN	PACK 690
BITOONE	DATA	0400000000000000000000		PACK 700
BITIONE	DATA	0200000000000000000000		PACK 710
	END			PACK 720

	SUBROUTINE REDUCE	REDU 10
	DIMENSION ISTD(1)	REDU 20
	DIMENSION IOBUF(444)	REDU 30
	COMMON / CMAIN /	REDU 40
1	RJXYZ (100,3), IJRFM (74,6), IJALFM(74,2), FRCVCT(74,3),	REDU 50
2	RMTVCT(74,3), IBARP (130), IBARQ (130), IBARR (130),	REDU 60
3	KS (130), BAREA (130), BARIN (130), BARIT (130),	REDU 70
4	BARJ (130), BARYM (130), BARSM (130), RKN (130),	REDU 80
5	RKT (130), SOLVEC(444), RLIMTU(88), RLIMTL(88),	REDU 90
6	DPRTIT(88), IROWL (445), STRSTF(1)	REDU 100
	DIMENSION IRWKP(103), SM(444), ROWNXT(444)	REDU 110
	DIMENSION T(444), IRCPST(444), IRWTOR(444)	REDU 120
	EQUIVALENCE (STRSTF(1), ISTD(1))	REDU 130
	EQUIVALENCE (FRCVCT(1,1), SM(1)), (BAREA(1), ROWNXT(1))	REDU 140
	EQUIVALENCE (SOLVEC(1) , IRWTOR(1)), (RLIMTL(15), IRWKP(1))	REDU 150
	EQUIVALENCE (IROWL(1), IOBUF(1))	REDU 160
	EQUIVALENCE (STRSTF(7964), IRCPST)	REDU 170
	EQUIVALENCE (STRSTF(9740), T)	REDU 180
	INTEGER TAPEA, TAPEB	REDU 190
	COMMON COM(30)	REDU 200
	EQUIVALENCE (COM (8), NROW)	REDU 210
	EQUIVALENCE (COM (22), IREDTO)	REDU 220
	EQUIVALENCE (COM (21), NRWTOR)	REDU 230
	EQUIVALENCE (COM (28), INDWTS)	REDU 240
		REDU 250
	DIMENSION ERSD(13)	REDU 255
	DIMENSION ERZD(12)	REDU 260
	DATA ERSD /50H TOO MANY ROWS HAVE BEEN REDUCED FROM THE STIFFNES	REDU 262
	1,60HS MATRIX LEADING TO A VERY SMALL DIAGONAL ELEMENT IN A ROW B,	REDU 264
	220HEING REMOVED /	REDU 266
	DATA ERZD /50H TOO MANY ROWS HAVE BEEN REDUCED FROM THE STIFFNES	REDU 270
	1,60HS MATRIX LEADING TO A ZERO DIAGONAL ELEMENT IN A ROW BEING R,	REDU 280
	210HEMOVED /	REDU 285
	TAPEA = 1	REDU 290
	TAPEB = 2	REDU 300
		REDU 310
	SET UP IRWTOR WITH THE ROWS TO BE REDUCED IN DESCENDING ORDER	REDU 320
		REDU 330
	IRWKP(IREDTO+1)=0	REDU 340
	J=1	REDU 350
	IRWTOR(1)=0	REDU 360
	IF (IREDTO.EQ.NROW) GO TO 30	REDU 370
	NRWTOR=NROW-IREDTO	REDU 380
	LAST=NRWTOR+1	REDU 390
	DO 20 I=1,NROW	REDU 400
	IF (I.EQ.IRWKP(J)) GO TO 10	REDU 410
	LAST=LAST-1	REDU 420
	IRWTOR(LAST)=I	REDU 430
	GO TO 20	REDU 440
10	J=J+1	REDU 450
20	CONTINUE	REDU 460
30	CONTINUE	REDU 470
		REDU 480
	WRITE STIFFNESS MATRIX ON TAPEA BY ROWS, WITH THE EXCEPTION	REDU 490
	THAT THE FIRST ROW TO BE REMOVED IS STORED IN ROWNXT	REDU 500

C		REWIND TAPEA	REDU 510
		L=1	REDU 520
		DO 160 I=1,NROW	REDU 530
		IF=1	REDU 540
40		L=L+1	REDU 550
		IF (ISTSTF(L).LT.0) GO TO 110	REDU 560
		IC=ISTSTF(L)	REDU 570
		IF (IC.GT.IF) GO TO 70	REDU 580
50		IL=IC+5	REDU 590
		DO 60 J=IF,IL	REDU 600
		L=L+1	REDU 610
		IF (ABS(STRSTF(L)).LT.1.E-8) STRSTF(L)=0.0	REDU 620
		IF (STRSTF(L).EQ.0.0) GO TO 60	REDU 630
		IF (INDWTS .NE. 0) CALL WRSTRK (I, J, STRSTF(L))	REDU 640
60		SM(J)=STRSTF(L)	REDU 650
		IF=IL+1	REDU 660
		GO TO 40	REDU 670
70		IL=IC-1	REDU 680
		IF (IF.EQ.0.OR.IL.EQ.0) GO TO 80	REDU 690
		IF (IABS(IF).LT.444.AND.IABS(IL).LT.444) GO TO 90	REDU 700
80		CONTINUE	REDU 710
		WRITE (6,340) IF,IL	REDU 720
		IF=L-200	REDU 730
		IL=L+200	REDU 740
		WRITE (6,350) (STRSTF(J),J=IF,IL)	REDU 750
		WRITE (6,360) (ISTSTF(J),J=IF,IL)	REDU 760
		STOP	REDU 770
90		CONTINUE	REDU 780
		DO 100 J=IF,IL	REDU 790
100		SM(J)=0.0	REDU 800
		IF=IC	REDU 810
		GO TO 50	REDU 820
110		CONTINUE	REDU 830
		IF (IF.GT.NROW) GO TO 130	REDU 840
		DO 120 J=IF,NROW	REDU 850
120		SM(J)=0.0	REDU 860
130		CONTINUE	REDU 870
		IF (I.NE.IRWTOR(1)) GO TO 150	REDU 880
		DO 140 J=1,NROW	REDU 890
140		ROWNXT(J)=SM(J)	REDU 900
		GO TO 160	REDU 910
C		PACK AND WRITE THE ROW IN BLOCK FORMAT	REDU 920
150		CALL PACK (SM(1), NROW, IOBUF(1))	REDU 930
		KWRIT = IOBUF(1)	REDU 940
		WRITE(TAPEA) KWRIT, (IOBUF(J),J=1,KWRIT)	REDU 950
160		CONTINUE	REDU 960
C		PRINT LAST LINE	REDU 970
		IF (INDWTS .NE. 0) CALL WRSTRK(-1,-1,1.)	REDU 980
		DO 170 J=1,NROW	REDU 990
170		IRCPST(J)=J	REDU1000
C			REDU1010
C		SET NUMBER OF ACTIVE ROWS	REDU1020
C			REDU1030
C			REDU1040
		NACTRW=NROW	REDU1050
		KREAD = 0	REDU1060

	KREAD1 = 1	REDU1070
	IF (IREDTO.EQ.NROW) GO TO 320	REDU1080
C		REDU1090
C	START OF LOOP TO REDUCE STRUCTURAL STIFFNESS MATRIX	REDU1100
C		REDU1110
	ICOUNT=0	REDU1112
	SAVTOT=0.	REDU1114
	KWRIT1= 4402	REDU1120
	KWRIT = 2502	REDU1130
	REWIND 9	REDU1140
	IRWTOR(NRWTOR+1)=0	REDU1150
	DO 310 IRWR=1,NRWTOR	REDU1160
	REWIND TAPEA	REDU1170
	REWIND TAPEB	REDU1180
	IROW=IRWTOR(IRWR)	REDU1190
	IR1=IRCPST(IROW)	REDU1200
	IF (IRWR.NE.NRWTOR) GO TO 180	REDU1210
C	REDUCE KWRIT1 TO FIT LATER BUFFER LIMITS	REDU1220
	KWRIT1= 2830	REDU1230
	IR2=0	REDU1240
C	IR1 CURRENT POSITION (ROW + COL. NO.) OF ROW BEING	REDU1250
C		REMOVED REDU1260
C	IR2 CURRENT POSITION OF NEXT ROW TO BE REMOVED	REDU1270
C		REDU1280
	GO TO 190	REDU1290
180	IR2=IRWTOR(IRWR+1)	REDU1300
	IR2=IRCPST(IR2)	REDU1310
190	CONTINUE	REDU1320
	IRR=IR1-1	REDU1330
C		REDU1340
C	BUILD TRANSFORMATION MATRIX OF ORDER (NACTRW)*(NACTRW-1)	REDU1350
C	SAVE ONLY LAST ROW, SINCE FIRST NACTRW-1 ROWS ARE THE	REDU1360
C	IDENTITY MATRIX	REDU1370
	NACTR2=NACTRW-1	REDU1380
	SAVE=ROWNXT(IR1)	REDU1390
	ICOUNT=ICOUNT+1	REDU1392
	SAVTOT = SAVTOT + ABS(SAVE)	REDU1394
	AVSAVE=SAVTOT/ICOUNT	REDU1396
	IF(SAVE.EQ.0.) CALL ERPNT1(ERZD,12,1)	REDU1400
	IF(ABS(SAVE)/AVSAVE.LT.1.E-06) CALL ERPNT1(ERSD,13,0)	REDU1405
	DO 200 I=1,IRR	REDU1410
200	T(I)=-ROWNXT(I)/SAVE	REDU1420
	IF (IR1 .GT. NACTR2) GO TO 212	REDU1430
	DO 210 I=IR1,NACTR2	REDU1440
210	T(I)=-ROWNXT(I+1)/SAVE	REDU1450
212	CONTINUE	REDU1460
C	*****	REDU1470
C		REDU1480
C	SAVE LAST ROW OF TRANSFORMATION MATRIX ON FILE UNIT 9	REDU1490
C		REDU1500
C	WRITE(9) (T(K), K=1,NACTR2)	REDU1510
C		REDU1520
C		REDU1530
C	START LOOP TO PROCESS ACTIVE ROW IN COORDINATE REDUCTION	REDU1540
C		REDU1550
C	DO 290 ICRNTR=1,NACTR2	REDU1560

C	READ STIFFNESS MATRIX ROW, ICRNTR	REDU1570
C	BREAK DOWN THE INPUT BLOCKS	REDU1580
	IF (KREAD .GT. KREAD1) GO TO 215	REDU1590
	KREAD1 = 1	REDU1600
	READ (TAPEA) KREAD,(STRSTF(I),I=1,KREAD)	REDU1610
215	CALL UNPACK (STRSTF(KREAD1), NACTRW, SM(1))	REDU1620
	KREAD1 = KREAD1 + ISTSTF(KREAD1)	REDU1630
C		REDU1640
C		REDU1650
	SAVE=SM(IR1)	REDU1660
	IF (SAVE.NE.0.) GO TO 230	REDU1670
	IF (IR1 .GT. NACTR2) GO TO 260	REDU1680
	DO 220 I=IR1,NACTR2	REDU1690
220	SM(I)=SM(I+1)	REDU1700
	GO TO 260	REDU1710
230	DO 240 I=1,IRR	REDU1720
240	SM(I)=SM(I)+SAVE*T(I)	REDU1730
	IF (IR1 .GT. NACTR2) GO TO 260	REDU1740
	DO 250 I=IR1,NACTR2	REDU1750
250	SM(I)=SM(I+1)+SAVE*T(I)	REDU1760
260	CONTINUE	REDU1770
C		REDU1780
C	IF THIS ROW IS THE NEXT TO BE REMOVED, SAVE THE ROW IN RCWNXT	REDU1790
C		REDU1800
	IF (ICRNTR.NE.IR2) GO TO 280	REDU1810
	DO 270 I=1,NACTR2	REDU1820
270	RCWNXT(I)=SM(I)	REDU1830
	IF (ICRNTR .NE. NACTR2) GO TO 290	REDU1840
	IF (KWRIT .NE. 2502) GO TO 285	REDU1850
	GO TO 290	REDU1860
C		REDU1870
C	WRITE THE REDUCED ROW ON TO TAPEB	REDU1880
C		REDU1890
C	BLOCK OUTPUT	REDU1900
280	CALL PACK (SM(1), NACTR2, STRSTF(KWRIT))	REDU1910
	KWRIT = KWRIT + ISTSTF(KWRIT)	REDU1920
	IF (ICRNTR .EQ. NACTR2) GO TO 285	REDU1930
	IF (KWRIT .LT.KWRIT1) GO TO 287	REDU1940
285	ISTSTF(2501) = KWRIT - 2502	REDU1950
	KWRIT = KWRIT - 1	REDU1960
	WRITE(TAPEB)(STRSTF(I), I= 2501, KWRIT)	REDU1970
	KWRIT = 2502	REDU1980
287	CONTINUE	REDU1990
C		REDU2000
290	CONTINUE	REDU2010
C		REDU2020
C	REVERSE READ/WRITE TAPE VALUES	REDU2030
C		REDU2040
	I=TAPEA	REDU2050
	TAPEA=TAPEB	REDU2060
	TAPEB=I	REDU2070
C	*****	REDU2080
C		REDU2090
C	UPDATE IRCPST ARRAY TO ADJUST FOR THE ROW + COLUMN REMOVED	REDU2100
C		REDU2110
	IRCPST(IROW)=-IRCPST(IROW)	REDU2120

	K=IROW+1	REDU2130
	DO 300 I=K,NROW	REDU2140
	IF (IRCPST(I).LT.0) GO TO 300	REDU2150
	IRCPST(I)=IRCPST(I)-1	REDU2160
300	CONTINUE	REDU2170
C		REDU2180
C	UPDATE NO. OF ACTIVE ROWS	REDU2190
C		REDU2200
	NACTRW=NACTR2	REDU2210
310	CONTINUE	REDU2220
C		REDU2230
C	COPY STIFFNESS MATRIX TO CORE	REDU2240
C		REDU2250
320	CONTINUE	REDU2260
	IRR=0	REDU2270
	REWIND TAPEA	REDU2280
	DO 330 I=1,NACTRW	REDU2290
C		REDU2300
C	BREAK DOWN THE INPUT BLOCKS	REDU2310
C		REDU2320
	IF (KREAD .GT. KREAD1) GO TO 325	REDU2330
	KREAD1 = 1	REDU2340
	READ (TAPEA) KREAD, (IOBUF(I), I=1, KREAD)	REDU2350
325	CALL UNPACK (IOBUF(KREAD1), NACTRW, SM(1))	REDU2360
	KREAD1 = KREAD1 + IOBUF(KREAD1)	REDU2370
	DO 330 J=1,NACTRW	REDU2380
	IRR=IRR+1	REDU2390
330	STRSTF(IRR)=SM(J)	REDU2400
C		REDU2410
C	REVERSE THE RECORD POSITION IN THE TRANSFORMATION	REDU2420
C	MATRIX FILE AND PLACE ON UNIT 1	REDU2430
C		REDU2440
	IF(IREDTO.EQ.NROW) RETURN	REDU2445
	REWIND 1	REDU2450
	NACTRW = IREDTO	REDU2460
	DO 335 I = 1, NRWTOR	REDU2470
	BACKSPACE 9	REDU2480
	READ(9)(SM(L),L=1, NACTRW)	REDU2490
	WRITE(1)(SM(L),L=1, NACTRW)	REDU2500
	BACKSPACE 9	REDU2510
	NACTRW = NACTRW + 1	REDU2520
335	CONTINUE	REDU2530
C		REDU2540
	RETURN	REDU2550
C		REDU2560
340	FORMAT (7H IF IL ,2I10)	REDU2570
350	FORMAT (10E12.5)	REDU2580
360	FORMAT (10I12)	REDU2590
	END	REDU2600

	SUBROUTINE STFMAS (STRSTF,AMASS,MORDER)	STFM 10
	DIMENSION STRSTF(MORDER,MORDER), AMASS(1)	STFM 20
C		STFM 30
C	COMPUTE (M)**-1/2 = ((M)**-1/2)T FOR DIAGONAL MATRIX	STFM 40
C		STFM 50
	DO 10 I=1,MORDER	STFM 60
10	AMASS(I)=1/SQRT(AMASS(I))	STFM 70
C		STFM 80
C	COMPUTE ((M)**-1/2)T * (K) * (M)**-1/2	STFM 90
C	STORE THE RESULT IN LOWER TRIANGLE INCLUDING DIAGONAL	STFM 100
C	FORMAT IN K	STFM 110
	DO 20 I=1,MORDER	STFM 120
	DO 20 J=I,MORDER	STFM 130
20	STRSTF(I,J)=STRSTF(I,J)*AMASS(J)	STFM 140
	LOCSTF=0	STFM 150
	DO 30 I=1,MORDER	STFM 160
	DO 30 J=1,I	STFM 170
	LOCSTF=LOCSTF+1	STFM 180
30	STRSTF(LOCSTF,1)=STRSTF(J,I)*AMASS(J)	STFM 190
	RETURN	STFM 200
	END	STFM 210

	SUBROUTINE FNALEV (EGVECT,AMASS,MORDER,NEIGVL,NROW,NRWTOR,T,T2,IRW	FNAL 10
	1KP,IRWTOR)	FNAL 20
	DIMENSION T2(1), IRWKP(1), IRWTOR(1)	FNAL 30
	DIMENSION AMASS(1), EGVECT(1), T(1)	FNAL 40
C		FNAL 50
C	CALCULATE EIGENVECTORS FOR THE NON-REDUCED MATRIX FROM	FNAL 60
C	TRANSFORMATION MATRICES STORED IN FILE 9 BY SUBROUTINE	FNAL 70
C	REDUCE	FNAL 80
C		FNAL 90
	REWIND 1	FNAL 100
	NACTRW=MORDER	FNAL 110
	DO 90 I=1,NRWTOR	FNAL 120
	READ (1) (T(L),L=1,NACTRW)	FNAL 130
	IF (I.EQ.1) GO TO 40	FNAL 140
	I4=1	FNAL 150
	I1=1	FNAL 160
	I2=NRWTOR	FNAL 170
	I3=MORDER+1	FNAL 180
	DO 30 J=1,NACTRW	FNAL 190
	IF (I4.EQ.I) GO TO 10	FNAL 200
	IF (IRWKP(I1).GT.IRWTOR(I2)) GO TO 20	FNAL 210
10	T2(I1)=T(J)	FNAL 220
	I1=I1+1	FNAL 230
	GO TO 30	FNAL 240
20	T2(I3)=T(J)	FNAL 250
	I2=I2-1	FNAL 260
	I3=I3+1	FNAL 270
	I4=I4+1	FNAL 280
30	CONTINUE	FNAL 290
	GO TO 60	FNAL 300
40	CONTINUE	FNAL 310
	DO 50 J=1,NACTRW	FNAL 320
50	T2(J)=T(J)	FNAL 330
60	CONTINUE	FNAL 340
	K=0	FNAL 350
	DO 80 J=1,5	FNAL 360
	SAVE=0.0	FNAL 370
	DO 70 M=1,NACTRW	FNAL 380
	L=K+M	FNAL 390
70	SAVE=SAVE+T2(M)*EGVECT(L)	FNAL 400
	EGVECT(L+1)=SAVE	FNAL 410
80	K=K+NROW	FNAL 420
	NACTRW=NACTRW+1	FNAL 430
90	CONTINUE	FNAL 440
	RETURN	FNAL 450
	END	FNAL 460

	SUBROUTINE EEPNT (EGVALU,EGVECT,NEIGVL,NROW)	EEPN 10
	DIMENSION EGVALU(1), EGVECT(1)	EEPN 20
C	K1=1	EEPN 30
	K2=NROW	EEPN 40
	DO 10 I=1,NEIGVL	EEPN 50
	WRITE (6,20) I,EGVALU(I)	EEPN 60
		EEPN 70
C	IF (I. .LT. 7) GO TO 40	EEPN 80
	WRITE (6,30) (EGVECT(K),K=K1,K2)	EEPN 90
	K1=K1+NROW	EEPN 100
	K2=K2+NROW	EEPN 110
10	CONTINUE	EEPN 120
	RETURN	EEPN 130
C		EEPN 140
20	FORMAT (13H0 EIGENVALUE(,I2,3H) =,E14.7)	EEPN 150
30	FORMAT (10X30H THE CORRESPONDING EIGENVECTOR/(10X10E12.4))	EEPN 160
	END	EEPN 170

	SUBROUTINE TRIDIA (NR,R,A,PQ)	TRID 10
C	TRI-DIAGONALIZES SYMMETRIC MATRIX BY HOUSEHOLDER METHOD	TRID 20
C	NR = ORDER OF MATRIX	TRID 30
C	R = 1-DIMENSIONAL ARRY OF NR/2*(NR+1) ELEMENTS	TRID 40
C	1. CONTAINS LOWER TRIANGULAR PART PLUS DIAGONAL OF MATRIX	TRID 50
C	2. AFTER TRIPLE-DIAGONALIZATION CONTAINS THE DIAGONAL PLUS	TRID 60
C	W VECTORS	TRID 70
C	A = 1-DIMENSIONAL ARRAY OF NR ELEMENTS, CONTAINS DIAGONAL	TRID 80
C	ELEMENTS OF TRIPLE-DIAGONAL MATRIX	TRID 90
C	PQ = 1-DIMENSIONAL ARRAY OF NR ELEMENTS,	TRID 100
C	1. CONTAINS ELEMENTS OF P VECTOR	TRID 110
C	2. CONTAINS ELEMENTS OF Q VECTOR	TRID 120
C	3. CONTAINS THE OFF-DIAGONAL TERM OF TRIPLE-DIAGONAL MATRIX	TRID 130
C	(END RESULT =(3))	TRID 140
C	DIMENSION R(1),PQ(1),A(1)	TRID 150
	IA=0	TRID 160
	NR1=NR-1	TRID 170
	DO 180 I=2,NR1	TRID 180
	IA=IA+I	TRID 190
C	CALCULATE ELEMENTS OF W VECTOR	TRID 200
	S=0.	TRID 210
	JA=IA	TRID 220
	DO 10 J=I,NR	TRID 230
	S=S+R(JA)**2	TRID 240
	JA=JA+J	TRID 250
10	CONTINUE	TRID 260
	SS=SQRT(S)	TRID 270
	PQ(I-1)=-SIGN(SS,R(IA))	TRID 280
	IF (S) 20,190,20	TRID 290
20	X=-PQ(I-1)/S	TRID 300
30	R(IA)=SQRT(.500*(1.00+R(IA)*X))	TRID 310
	IF (S) 50,40,50	TRID 320
40	R(IA)=0.	TRID 330
50	IF (R(IA)) 60,200,60	TRID 340
60	X=.500*X/R(IA)	TRID 350
70	JA=IA	TRID 360
	DO 80 J=I,NR1	TRID 370
	JA=JA+J	TRID 380
	R(JA)=X*R(JA)	TRID 390
80	CONTINUE	TRID 400
C	CALCULATE ELEMENTS OF P VECTOR	TRID 410
	JAI=IA+1	TRID 420
	DO 130 J=I,NR	TRID 430
	JA=JAI	TRID 440
	KA=IA	TRID 450
	PQ(J)=0.	TRID 460
	DO 120 K=I,NR	TRID 470
	PQ(J)=PQ(J)+R(KA)*R(JA)	TRID 480
	IF (K-J) 90,100,100	TRID 490
90	JA=JA+1	TRID 500
	GO TO 110	TRID 510
100	JA=JA+K	TRID 520
110	KA=KA+K	TRID 530
120	CONTINUE	TRID 540
	JAI=JAI+J	TRID 550

130	CONTINUE	TRID 560
C	CALCULATE ELEMENTS OF Q VECTOR	TRID 570
	AK=0.	TRID 580
	JA=IA	TRID 590
	DO 140 J=I,NR	TRID 600
	AK=AK+R(JA)*PQ(J)	TRID 610
	JA=JA+J	TRID 620
140	CONTINUE	TRID 630
	AK=2.00*AK	TRID 640
	JA=IA	TRID 650
	DO 150 J=I,NR	TRID 660
	PQ(J)=2.00*PQ(J)-AK*R(JA)	TRID 670
	JA=JA+J	TRID 680
150	CONTINUE	TRID 690
C	CALCULATE ELEMENTS OF NEW R MATRIX	TRID 700
	KK=IA	TRID 710
	JA=IA	TRID 720
	DO 170 J=I,NR	TRID 730
	KA=IA	TRID 740
	DO 160 K=I,J	TRID 750
	KK=KK+1	TRID 760
	R(KK)=R(KK)-PG(K)*R(JA)-R(KA)*PQ(J)	TRID 770
	KA=KA+K	TRID 780
160	CONTINUE	TRID 790
	KK=KK+I-1	TRID 800
	JA=JA+J	TRID 810
170	CONTINUE	TRID 820
180	CONTINUE	TRID 830
	GO TO 210	TRID 840
190	X=0.	TRID 850
	GO TO 30	TRID 860
200	X=0.	TRID 870
	GO TO 70	TRID 880
C	SORT ALPHAS AND BETAS	TRID 890
210	IA=0	TRID 900
	DO 220 I=1,NR	TRID 910
	IA=IA+I	TRID 920
	A(I)=R(IA)	TRID 930
220	CONTINUE	TRID 940
	PQ(NR)=R(IA-1)	TRID 950
	N=NR	TRID 960
230	N=N-1	TRID 970
	PQ(N)=PQ(N-1)	TRID 980
	IF (2-N) 230,240,240	TRID 990
240	PQ(1)=0.	TRID1000
	RETURN	TRID1010
	END	TRID1020

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SUBROUTINE EIGVAL (LP,E,A,B,F,W,MVAL)                                EIVL 10
C FINDS EIGENVALUES OF TRI-DIAGONAL MATRICES BY ORTEGA METHOD      EIVL 20
C   LP = ORDER OF MATRIX                                          EIVL 30
C   E = 1-DIMENSIONAL ARRAY OF LP ELEMENTS. THE EIGENVALUFS ARE EIVL 40
C   STORED IN THIS ARRAY IN ASCENDING ORDER                      EIVL 50
C   A = 1-DIMENSIONAL ARRAY OF LP ELEMENTS. CONTAINS DIAGONAL   EIVL 60
C   ELEMENTS OF TRIPLE-DIAGONAL MATRIX                          EIVL 70
C   B = 1-DIMENSIONAL ARRAY OF LP ELEMENTS. CONTAINS OFF-DIAGONAL EIVL 80
C   ELEMENTS OF TRIPLE-DIAGONAL MATRIX                          EIVL 90
C   F AND W = 1-DIMENSION ARRAY OF LP ELEMENTS EACH USED AS    EIVL 100
C   WORK AREAS                                                  EIVL 110
C   MVAL = NUMBER OR EIGENVALUES TO BE CALCULATED.              EIVL 120
DIMENSION E(1),A(1),B(1),F(1),W(1)                                EIVL 130
C FIND UPPER AND LOWER BOUNDS AND NORMALIZE INPUT                EIVL 140
BD=ABS(A(1))                                                       EIVL 150
DO 10 I=2,LP                                                       EIVL 160
10 BD=AMAX1(BD,ABS(A(I))+B(I)**2,0.00)                             EIVL 170
   BD=BD+1.                                                         EIVL 180
   DO 20 I=1,LP                                                    EIVL 190
   A(I)=A(I)/BD                                                    EIVL 200
   B(I)=B(I)/BD                                                    EIVL 210
   W(I)=1.                                                         EIVL 220
20 E(I)=-1.                                                         EIVL 230
   DO 230 K=1,MVAL                                                EIVL 240
30 IF ((W(K)-E(K))/AMAX1(ABS(W(K)),ABS(E(K)),1.0E-9)-1.0E-12) 230,230 EIVL 250
1,40                                                                EIVL 260
40 X=(W(K)+E(K))*0.5                                               EIVL 270
C   FIND NUMBER OF EIGENVALUES, N, GREATER THAN OR EQUAL TO X   EIVL 280
   S2=1.                                                           EIVL 290
   IS2=1                                                           EIVL 300
   F(1)=A(1)-X                                                     EIVL 310
   IF (F(1)) 50,60,60                                             EIVL 320
50 IS1=-1                                                         EIVL 330
   S1=-1.                                                         EIVL 340
   N=0                                                             EIVL 350
   GO TO 70                                                       EIVL 360
60 IS1=1                                                           EIVL 370
   S1=1.                                                           EIVL 380
   N=1                                                             EIVL 390
70 DO 170 I=2,LP                                                  EIVL 400
   IF (B(I)) 80,120,80                                            EIVL 410
80 IF (B(I-1)) 90,130,90                                          EIVL 420
90 IF (ABS(F(I-1))+ABS(F(I-2))-1.0E-15) 100,110,110            EIVL 430
100 F(I-1)=F(I-1)*1.0E15                                          EIVL 440
   F(I-2)=F(I-2)*1.0E15                                          EIVL 450
110 F(I)=(A(I)-X)*F(I-1)-B(I)**2*F(I-2)                          EIVL 460
   GO TO 140                                                      EIVL 470
120 F(I)=(A(I)-X)*SIGN(1.00,S1)                                    EIVL 480
   GO TO 140                                                      EIVL 490
130 F(I)=(A(I)-X)*F(I-1)-SIGN(P(I)**2,S2)                        EIVL 500
140 S2=S1                                                         EIVL 510
   IS2=IS1                                                         EIVL 520
   IF (F(I)) 150,160,150                                          EIVL 530
150 S1=SIGN(ABS(S1),F(I))                                         EIVL 540
   IS1=S1                                                         EIVL 550

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	IF (IS2+IS1) 160,170,160	EIVL 560
160	N=N+1	EIVL 570
170	CONTINUE	EIVL 580
C	TRAP EIGENVALUES IN SMALLER AND SMALLER BOUNDS	EIVL 590
	N=LP-N	EIVL 600
	IF (N-K) 200,180,180	EIVL 610
180	DO 190 J=K,N	EIVL 620
190	W(J)=X	EIVL 630
200	N=N+1	EIVL 640
	IF (LP-N) 30,210,210	EIVL 650
210	DO 220 J=N,LP	EIVL 660
	IF (X-E(J)) 30,30,220	EIVL 670
220	E(J)=X	EIVL 680
	GO TO 30	EIVL 690
230	CONTINUE	EIVL 700
C	RESTORE INPUT AND ORDER EIGENVALUES	EIVL 710
	DO 240 I=1,LP	EIVL 720
	A(I)=A(I)*BD	EIVL 730
240	B(I)=B(I)*BD	EIVL 740
	DO 250 I=1,MVAL	EIVL 750
250	E(I)=(W(I)+E(I))*BD*.5	EIVL 760
	RETURN	EIVL 770
	END	EIVL 780

	SUBROUTINE EIGVEC (NR,E,R,A,B,V,P,Q)	EIVC 10
C	GIVEN AN EIGENVALUE, FIND THE CORRESPONDING EIGENVECTOR USING	EIVC 20
C	WILKINSON METHOD	EIVC 30
C	NR = ORDER OF MATRIX	EIVC 40
C	R = 1-DIMENSION ARRAY OF NR/2*(NR+1) ELEMENTS CONTAINING	EIVC 50
C	THE W VECTOR	EIVC 60
C	E = EIGENVALUE	EIVC 70
C	A = 1-DIMENSION ARRAY OF NR ELEMENTS CONTAINING DIAGONAL	EIVC 80
C	TERMS OF TRIPLE-DIAGONAL MATRIX	EIVC 90
C	B = 1-DIMENSION ARRAY OF NR ELEMENTS CONTAINING OFF-DIAGONAL	EIVC 100
C	TERMS OF TRIPLE-DIAGONAL MATRIX	EIVC 110
C	V = 1-DIMENSION ARRAY OF NR ELEMENTS CONTAINING THE	EIVC 120
C	EIGENVECTOR OF THE ORIGINAL SYMMETRIC MATRIX	EIVC 130
C	P AND Q = 1-DIMENSIONAL ARRAYS OF NR ELEMENTS EACH USED AS	EIVC 140
C	WORK AREAS	EIVC 150
	DIMENSION R(1),A(1),B(1),V(1),P(1),Q(1)	EIVC 160
	NR1=NR-1	EIVC 170
C	SET UP SIMULTANEOUS EQUATIONS I.E. COMPUTE P, Q, R	EIVC 180
	X=A(1)-E	EIVC 190
	Y=B(2)	EIVC 200
	DO 70 I=1,NR1	EIVC 210
	IF (ABS(B(I+1))-ABS(X)) 30,10,50	EIVC 220
10	IF (X) 30,20,30	EIVC 230
20	X=1.0E-10	EIVC 240
30	P(I)=X	EIVC 250
	Q(I)=Y	EIVC 260
	V(I)=0.	EIVC 270
	X=A(I+1)-E-B(I+1)*Y/X	EIVC 280
	IF (NR1-I) 40,70,40	EIVC 290
40	Y=B(I+2)	EIVC 300
	GO TO 70	EIVC 310
50	P(I)=B(I+1)	EIVC 320
	Q(I)=A(I+1)-E	EIVC 330
	Z=X/P(I)	EIVC 340
	X=Y-Z*Q(I)	EIVC 350
	IF (NR1-I) 60,70,60	EIVC 360
60	V(I)=B(I+2)	EIVC 370
	Y=-Z*V(I)	EIVC 380
70	CONTINUE	EIVC 390
C	SOLVE FOR EIGENVECTOR OF TRI-DIAGONAL MATRIX	EIVC 400
	IF (X) 90,80,90	EIVC 410
80	V(NR)=1.E10	EIVC 420
	GO TO 100	EIVC 430
90	V(NR)=1.00/X	EIVC 440
100	I=NR1	EIVC 450
	V(I)=(1.00-Q(I)*V(NR))/P(I)	EIVC 460
	X=V(NR)**2+V(I)**2	EIVC 470
110	I=I-1	EIVC 480
	IF (I) 120,130,120	EIVC 490
120	V(I)=(1.00-Q(I)*V(I+1)-V(I)*V(I+2))/P(I)	EIVC 500
	X=X+V(I)**2	EIVC 510
	GO TO 110	EIVC 520
C	NORMALIZE VECTOR	EIVC 530
130	X=SQRT(X)	EIVC 540
	DO 140 I=1,NR	EIVC 550

	V(I)=V(I)/X	EIVC 560
140	CONTINUE	EIVC 570
C	COMPUTE EIGENVECTOR OF ORIGINAL MATRIX	EIVC 580
	I=NR	EIVC 590
150	I=I-1	EIVC 600
	J=0	EIVC 610
	K=1	EIVC 620
160	K=K+1	EIVC 630
	J=J+K	EIVC 640
	IF (K-I) 160,170,160	EIVC 650
170	Y=0.	EIVC 660
	JA=J	EIVC 670
	DO 180 K=I, NR	EIVC 680
	Y=Y+R(JA)*V(K)	EIVC 690
	JA=JA+K	EIVC 700
180	CONTINUE	EIVC 710
	Y=2.00*Y	EIVC 720
	JA=J	EIVC 730
	DO 190 K=I, NR	EIVC 740
	V(K)=V(K)-Y*R(JA)	EIVC 750
	JA=JA+K	EIVC 760
190	CONTINUE	EIVC 770
	IF (I-2) 150,200,150	EIVC 780
200	RETURN	EIVC 790
	END	EIVC 800

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SUBROUTINE PRNT1 (EGVALU,EGVECT,AMASS,IRWKP,NROW,MORDER,IRWTOR,WORPRNT 10
1KA) PRNT 20
DIMENSION WORKA(1) PRNT 30
DIMENSION EGVALU(1), EGVECT(NROW,5), AMASS(1), IRWKP(1), IRWTOR(1) PRNT 40
COMMON COM(30) PRNT 50
EQUIVALENCE ( COM( 1 ), NJOINT ) PRNT 60
EQUIVALENCE ( COM(25), INDWNM ) PRNT 70
COMMON / CMAIN / COMAIN(1) PRNT 80
EQUIVALENCE ( COMAIN( 1), RJX ) PRNT 90
EQUIVALENCE ( COMAIN( 101), RJY ) PRNT 100
EQUIVALENCE ( COMAIN( 201), RJZ ) PRNT 110
DIMENSION WNX(5), WNY(5), WNZ(5), PX(5), PY(5), PZ(5), GM(5) PRNT 120
1 , RJX(1), RJY(1), RJZ(1) PRNT 130
DIMENSION ALPHA(6) PRNT 140
DIMENSION IGVNO(5) PRNT 150
DATA ALPHA / 8HX-TRANS., 8HY-TRANS., 8HZ-TRANS. , PRNT 160
1 8HX-ROTAT., 8HY-ROTAT., 8HZ-ROTAT. / PRNT 170
DATA ITAPE / 9 / PRNT 180
IF ( IFRIST .NE. 0 ) GO TO 4 PRNT 190
IFRIST = 1 PRNT 200
WRITE (6,280) (EGVALU(I),I=1,6) PRNT 210
C SETUP EIGENVECTOR NO. SYMBOLS PRNT 220
DO 2 I = 1, 5 PRNT 230
2 IGVNO(I) = I PRNT 240
LR = 2 PRNT 250
MR = 6 PRNT 260
GO TO 6 PRNT 270
4 CONTINUE PRNT 280
C UPDATE EIGENVECTOR NO. SYMBOLS PRNT 290
DO 8 I = 1, 5 PRNT 300
8 IGVNO(I) = IGVNO(I) + 5 PRNT 310
6 CONTINUE PRNT 320
LR = LR + 5 PRNT 330
MR = MR + 5 PRNT 340
DO 10 J = LR, MR PRNT 350
10 EGVALU(J)=SQRT(EGVALU(J)) PRNT 360
WRITE(6,290) (IGVNO(I),I=1,5), (EGVALU(I),I=LR,MR) PRNT 370
IF (NROW.EQ.MORDER) GO TO 20 PRNT 380
WRITE (6,300) PRNT 390
GO TO 40 PRNT 400
20 DO 30 I=1,NROW PRNT 410
30 IRWKP(I)=I PRNT 420
WRITE (6,400) PRNT 430
40 CONTINUE PRNT 440
WRITE (6,310) PRNT 450
LCOUNT=11 PRNT 460
C PRNT 470
C ZERO OUT SUMMING FIELDS PRNT 480
C PRNT 490
DO 50 I=1,5 PRNT 500
WNX(I)=0.0 PRNT 510
WNY(I)=0.0 PRNT 520
WNZ(I)=0.0 PRNT 530
PX(I)=0.0 PRNT 540
PY(I)=0.0 PRNT 550

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	PZ(I)=0.0	PRNT 560
50	GM(I)=0.0	PRNT 570
C		PRNT 580
C	CALCULATE ABOVE VARIABLES	PRNT 590
C		PRNT 600
	N = (IRWKP(1)-1)/6+1	PRNT 610
	DO 140 I=1,MORDER	PRNT 620
	M=IRWKP(I)	PRNT 630
	K=M-(M-1)/6*6	PRNT 640
	M=(M-1)/6+1	PRNT 650
	IF (LCOUNT.LT.56) GO TO 60	PRNT 660
	WRITE (6,320)	PRNT 670
	IF(NROW.EQ.MORDER)GO TO 500	PRNT 675
	WRITE (6,300)	PRNT 680
	GO TO 501	PRNT 682
500	WRITE(6,400)	PRNT 684
501	CONTINUE	PRNT 686
	WRITE(6,310) (IGVNO(I),I=1,5)	PRNT 690
	LCOUNT=5	PRNT 700
	N=M	PRNT 710
	GO TO 70	PRNT 720
60	CONTINUE	PRNT 730
	IF (N.EQ.M) GO TO 70	PRNT 740
	WRITE (6,330)	PRNT 750
	LCOUNT=LCOUNT+1	PRNT 760
	N=M	PRNT 770
70	CONTINUE	PRNT 780
C		PRNT 790
C	WRITE MODE SHAPES OF REDUCED SYSTEM	PRNT 800
C		PRNT 810
	WRITE (6,340) M,ALPHA(K),(EGVECT(I,J),J=1,5)	PRNT 820
	LCOUNT=LCOUNT+1	PRNT 830
	IF (K.GT.3) GO TO 140	PRNT 840
	IF (K-2) 80,100,120	PRNT 850
C	X TRANSLATION MODE SHAPE ELEMENT	PRNT 860
80	DO 90 J=1,5	PRNT 870
	SAVE=EGVECT(I,J)*AMASS(I)	PRNT 880
	PX(J)=PX(J)+RJX(M)*SAVE	PRNT 890
	SAVE=SAVE*EGVECT(I,J)	PRNT 900
	WNY(J)=WNY(J)+SAVE	PRNT 910
90	WNZ(J)=WNZ(J)+SAVE	PRNT 920
	GO TO 140	PRNT 930
C	Y TRANSLATION MODE SHAPE ELEMENT	PRNT 940
100	DO 110 J=1,5	PRNT 950
	SAVE=EGVECT(I,J)*AMASS(I)	PRNT 960
	PY(J)=PY(J)+RJY(M)*SAVE	PRNT 970
	SAVE=SAVE*EGVECT(I,J)	PRNT 980
	WNX(J)=WNX(J)+SAVE	PRNT 990
110	WNZ(J)=WNZ(J)+SAVE	PRNT1000
	GO TO 140	PRNT1010
C	Z TRANSLATION MODE SHAPE ELEMENT	PRNT1020
120	DO 130 J=1,5	PRNT1030
	SAVE=EGVECT(I,J)*AMASS(I)	RRNT1040
	PZ(J)=PZ(J)+RJZ(M)*SAVE	PRNT1050
	SAVE=SAVE*EGVECT(I,J)	PRNT1060
	WNX(J)=WNX(J)+SAVE	PRNT1070

130	WNY(J)=WNY(J)+SAVE	PRNT1080
140	CONTINUE	PRNT1090
	DO 150 I=1,5	PRNT1100
	DO 150 K=1,MORDER	PRNT1110
	GM(I) = GM(I) + AMASS(K)*EGVECT(K,I)**2	PRNT1120
150	CONTINUE	PRNT1130
	WRITE (6,350)	PRNT1140
	WRITE(6,360) (IGVNO(I),I=1,5)	PRNT1150
	WRITE (6,370) (WNY(I),I=1,5),(WZ(I),I=1,5),(PX(I),	PRNT1160
	I=1,5),(PY(I),I=1,5),(PZ(I),I=1,5), (GM(I),I=1,5)	PRNT1170
	RETURN	PRNT1180
C		PRNT1190
C	PRINT MODE SHAPES FOR COMPLETE SYSTEM	PRNT1200
C		PRNT1210
	ENTRY PRNT2	PRNT1220
	IF ((NROW.EQ.MORDER).AND.(INDWNM.EQ.0)) RETURN	PRNT1230
	IF (INDWNM.EQ.0) GO TO 170	PRNT1240
	SAVE=NJOINT	PRNT1250
	SAVE1=MORDER	PRNT1260
	WRITE (ITAPE,380) SAVE,SAVE1	PRNT1270
	DO 160 I=1,MORDER	PRNT1280
160	WORKA(I)=IRWKP(I)	PRNT1290
	CALL P1A721 (121,WORKA,MORDER,ITAPE)	PRNT1300
	WRITE (ITAPE,390) (RJY(I),RJZ(I),RJX(I),I ,I=1,NJOINT)	PRNT1310
	LC=0	PRNT1320
170	CONTINUE	PRNT1330
	IF (NROW.EQ.MORDER) GO TO 180	PRNT1340
	WRITE(6,290) (IGVNO(I),I=1,5), (EGVALU(I),I=LR,MR)	PRNT1350
	WRITE (6,400)	PRNT1360
	WRITE(6,310) (IGVNO(I),I=1,5)	PRNT1370
	LCOUNT=11	PRNT1380
180	CONTINUE	PRNT1390
	M=NROW-MORDER	PRNT1400
	K=1	PRNT1410
	N=0	PRNT1420
	DO 260 I=1,NJOINT	PRNT1430
	DO 250 J=1,3	PRNT1440
	N=N+1	PRNT1450
190	IF (N.NE.IRWKP(K)) GO TO 200	PRNT1460
	N1=K	PRNT1470
	K=K+1	PRNT1480
	GO TO 220	PRNT1490
200	IF (N.NE.IRWTOR(M)) GO TO 210	PRNT1500
	N1=NROW-M+1	PRNT1510
	M=M-1	PRNT1520
	GO TO 220	PRNT1530
210	CONTINUE	PRNT1540
	IF (N.GT.IRWKP(K)) K=K+1	PRNT1550
	IF (N.GT.IRWTOR(M)) M=M-1	PRNT1560
	GO TO 190	PRNT1570
220	CONTINUE	PRNT1580
	IF (INDWNM.EQ.0) GO TO 230	PRNT1590
	LC=LC+1	PRNT1600
	WRITE (ITAPE,410) (EGVECT(N1,L),L=1,5),LC	PRNT1610
230	CONTINUE	PRNT1620
	IF (NROW.EQ.MORDER) GO TO 250	PRNT1630

```

WRITE (6,340) I,ALPHA(J),(EGVECT(N1,L),L=1,5)
LCOUNT=LCOUNT+1
IF (LCOUNT.LT.56) GO TO 240
WRITE (6,320)
WRITE (6,400)
WRITE(6,310) (IGVNO(I),I=1,5)
LCOUNT=5
GO TO 250
240 CONTINUE
IF (J.NE.3) GO TO 250
WRITE (6,330)
LCOUNT=LCOUNT+1
250 CONTINUE
N=N+3
260 CONTINUE
IF (INDWNM.EQ.0) GO TO 270
LR1 = LR - 1
WRITE (ITAPE,420) (EGVALU(LR1+I),GM(I ),I,I=1,5)
WRITE (ITAPE,430) (WNX(I),WNY(I),WNZ(I),PX(I),PY(I),PZ(I),I,I=1,5)
CALL PIA721 (721,AMASS,MORDER,ITAPE)
END FILE ITAPE
270 CONTINUE
RETURN
C
C
C INITIALIZE FOR EACH EIGENVALUE GROUP
C
ENTRY PRNT0
IFRIST = 0
RETURN
280 FORMAT (45H1EIGENVALUES ASSOCIATED WITH RIGID BODY MODES//(5XE14.7
1))
290 FORMAT (1H1,41X33HNATURAL FREQUENCIES (RADIAN/SEC)//
1 20X5I18//27X5E18.8//)
300 FORMAT (36X44HCORRESPONDING MODE SHAPES FOR REDUCED SYSTEM)
310 FORMAT (24H0 NODAL POINT ELEMENT/25H NUMBER REFERENCE,
1 11XI2, 4I18/ )
320 FORMAT (1H1)
330 FORMAT (1H )
340 FORMAT (6XI3,8XA8,2X5E18.8)
350 FORMAT (1H1,42X30HGENERALIZED INERTIA PROPERTIES/)
360 FORMAT ( 11H VARIABLES,9X5I18/)
370 FORMAT (1H04X,3HWNX,19X5E18.8/1H04X,3HWNY,19X5E18.8/1H04X,3HWNZ,19
1X5E18.8/1H04X,3H PX,19X5E18.8/1H04X,3H PY,19X5E18.8/1H04X,3H PZ,19
2X5E18.8/1H05X,15HGM DIAGONAL ,6X5E18.8/(27X5E18.8))
380 FORMAT (6X2H21,2X10H 5.0 ,2F10.3,37X3H 1)
390 FORMAT (5X3H221,2X3E10.3,37XI3)
400 FORMAT (36X45HCORRESPONDING MODE SHAPES FOR COMPLETE SYSTEM)
410 FORMAT (5X3H3212X,5E10.3,17XI3)
420 FORMAT (5X3H4212X,2E10.3,47XI3)
430 FORMAT (5X3H5212X,6E10.3,7XI3)
END

```

	SUBROUTINE P1A721 (NO,ARRAY,LANGTH,I6)	P1A7 10
	DIMENSION ARRAY(1)	P1A7 20
	K=1	P1A7 30
	I=1	P1A7 40
	DO 10 J=6,LANGTH,6	P1A7 50
	WRITE (I6,20) NO,(ARRAY(L),L=I,J),K	P1A7 60
	K=K+1	P1A7 70
	I=I+6	P1A7 80
10	CONTINUE	P1A7 90
	IF (I.GT.LANGTH) RETURN	P1A7 100
	WRITE (I6,20) NO,(ARRAY(L),L=I,LANGTH),K	P1A7 110
	RETURN	P1A7 120
C		P1A7 130
20	FORMAT (5X,I3,2X,6E10.3,7X I3)	P1A7 140
	END	P1A7 150

	OVERLAY (GEARRT,5,0)	OV50 10
	PROGRAM GEAREX	GEAR 10
	COMMON / CMAIN / COMSAP(2000)	GEAR 20
	EQUIVALENCE (COMSAP(1), LGRTYP)	GEAR 30
C	I/O UNIT 5 EQUALS INPUT UNIT	GEAR 40
C	I/O UNIT 6 EQUALS OUTPUT UNIT	GEAR 50
C	I/O UNIT 1 EQUALS WORK AREA--V.R.	GEAR 60
C	ZERO OUT COMMON	GEAR 70
	DO 150 I = 1, 2000	GEAR 80
	150 COMSAP(I) = 0.0	GEAR 90
C		GEAR 100
C	CALL THE INPUT ROUTINE	GEAR 110
C		GEAR 120
	CALL OVERLAY (6LGEARRT,5,1,6HRECALL)	GEAR 130
	IF(LGRTYP .EQ. 2) GO TO 200	GEAR 140
C		GEAR 150
C	CALL LANDING GEAR ANALYSIS ROUTINE FOR INVERTED TRIPOD GFAR	GEAR 160
C		GEAR 170
	CALL OVERLAY (6LGEARRT,5,2,6HRECALL)	GEAR 180
	GO TO 300	GEAR 190
	200 CONTINUE	GEAR 200
C		GEAR 210
C	CALL LANDING GEAR ANALYSIS ROUTINE FOR CANTILEVER GEAR	GEAR 220
C		GEAR 230
	CALL OVERLAY (6LGEARRT,5,3,6HRECALL)	GEAR 240
	300 CONTINUE	GEAR 250
	RETURN	GEAR 260
	END	GEAR 270

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SUBROUTINE OUTPUT
COMMON / CMAIN /
1  LGRTYP, LDIND , INITER, NPOINT, NREFP , NELEM , NUMB ,      OUTP 10
2  NSTEP , NITER , TOL , ENGABS, IFPATN, NFPCRL, FPCRLD(3),    OUTP 20
3  IGLDIR, NMATL , IST , IMATL(4) , MNO , FPCRST(3),          OUTP 30
4  INDPL(4) , ELASBN(4) , ELASAX(4) ,                          OUTP 40
5  BINERT(4) , AREA(4) , IRET(4) ,                             OUTP 50
6  IFIRST(4) , SCMAX(4) , STMAX(4) ,                           OUTP 60
7  SRULC(4) , SRULT(4), CDC(5,4), SRC(5,4) ,                    OUTP 70
8  CDT(5,4) , CRC(5,4) , SRT(5,4) ,                            OUTP 80
9  PFC(5,4) , PFT(5,4) , NPN , XYZPOS(3,5)                      OUTP 90
9  , DC(3,3) , PHI , THTA , PSI                                  OUTP 100
9  , CGYCDL(3), EGYCDL(3)                                       OUTP 110
COMMON / CMAIN /
A  NITERS, NDISPS, STLN2 ,                                       OUTP 120
B  IDSPDR, IDSPD1, IDSPD2, SPDISP, COEFF , XYZDSP(3),          OUTP 130
C  STLN(4) , STLNO(4) , STRP(4) ,                                OUTP 140
D  SPNGC(4) , SPNGT(4) , DISTC(4) ,                             OUTP 150
E  DISTT(4) , PFORC (4) , PFORT(4) ,                            OUTP 160
F  FOREVC(4) , FOREVT(4) , IPOSC(4) ,                           OUTP 170
G  IPOST(4) , INDULC(4) , INDULT(4) ,                           OUTP 180
H  STR(4) , AXIALF(4) , AXSTIF(4) ,                             OUTP 190
I  IPREV(4) , XYZDS (3) , SFORCE(30) ,                          OUTP 200
J  CFORCE(30) , SUMFM4(6) , R24XYZ(3) ,                         OUTP 210
K  TM(36) , FMVCTL(12,4) , FMVCTG(12,4) ,                       OUTP 220
L  YSHRFP(4), ZSHRFP(4), YSHRMP(4), ZSHRMP(4), CGYXYZ(3),      OUTP 230
M  YSHRFQ(4), ZSHRFQ(4), YSHRMQ(4), ZSHRMQ(4), EGYXYZ(3)      OUTP 240
DATA IPCNT, IPAGCT / 99, 0 /
IF ( IPCNT .LT. 2 ) GO TO 700
IPCNT = 0.0
IPAGCT = IPAGCT + 1
WRITE(6,799)IPAGCT
799 FORMAT ( 1H1, 123X 4HPAGE, I4 )
GO TO 570
700 CONTINUE
WRITE(6,760)
760 FORMAT ( 1H0, 33X6(10H***** ) )
570 CONTINUE
IPCNT = IPCNT + 1
NDISPS = NDISPS + 1
WRITE(6,800)NDISPS, NITERS
800 FORMAT ( 1H0,53X17H INCREMENTAL STEP,I5 /
1 38XI4,47H ITERATIONS REQUIRED FOR TOLERANCE SATISFACTION / )
ENTRY OUTPT1
N = 1
WRITE(6, 999)
999 FORMAT ( 8H ,13X22HCOORDINATE DEFINITIONS, 20X
1 15HEXTERNAL FORCES , 24X
2 16HEXTERNAL MOMENTS / 8H NODAL
2 4X 3( 7X25HSURFACE COORDINATE SYSTEM, 7X) /
3 3X5HPOINT, 4X3(6X1HX,12X1HY,12X1HZ,6X), / )
DO 210 I = 1, NPOINT
IF ( I .EQ. 4 ) GO TO 200
WRITE(6,1000)I,(XYZPOS(J,I),J=1,3),(FMVCTG(J,N),J=1,3 ),
1 (FMVCTG(J,N),J=7,9)
OUTP 250
OUTP 260
OUTP 270
OUTP 280
OUTP 290
OUTP 300
OUTP 310
OUTP 320
OUTP 330
OUTP 340
OUTP 350
OUTP 360
OUTP 370
OUTP 380
OUTP 390
OUTP 400
OUTP 410
OUTP 420
OUTP 430
OUTP 440
OUTP 450
OUTP 460
OUTP 470
OUTP 480
OUTP 490
OUTP 500
OUTP 510
OUTP 520
OUTP 530
OUTP 540
OUTP 550

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      N = N + 1                                OUTP 560
      GO TO 210                                OUTP 570
200  CONTINUE                                  OUTP 580
      WRITE(6,1000)I,(XYZPOS(J,I),J=1,3),(SUMFM4(J),J=1,6) OUTP 590
210  CONTINUE                                  OUTP 600
1000 FORMAT (4XI2,4X9E13.5)                   OUTP 610
      N = 1                                     OUTP 620
      WRITE(6,1099)                             OUTP 630
1099 FORMAT (20H0STRUCTURAL  AXIAL, 7X7H AXIAL , 7X5HAXIAL, OUTP 640
*      20X5HSHEAR,                             OUTP 650
1      8X5HSHEAR, 21X5HSHEAR, 8X5HSHEAR /      OUTP 660
2      48H ELEMENT FORCE STROKE STIFFNESS, 18X OUTP 670
3      5HFORCE, 8X6HMOMENT, 20X5HFORCE, 8X6HMOMENT / ) OUTP 680
      DO 220 I = 1, NELEM                       OUTP 690
C      ONE LOCAL SHEAR FORCE/MOMENT IS ZERO     OUTP 700
      SHARFP = YSHRFP(I) + ZSHRFP(I)           OUTP 710
      SHARMP = YSHRMP(I) + ZSHRMP(I)           OUTP 720
      SHARFQ = YSHRFQ(I) + ZSHRFQ(I)           OUTP 730
      SHARMQ = YSHRMQ(I) + ZSHRMQ(I)           OUTP 740
      J = I                                     OUTP 750
      IF ( I .EQ. 4 ) J = 5                     OUTP 760
220  WRITE(6,1100)I,AXIALF(I),STR(I) ,AXSTIF(I),J,SHARFP,SHARMP, OUTP 770
1      SHARFQ,SHARMQ                           OUTP 780
1100 FORMAT ( 4XI2,4X3E13.5,3X4HNODE I2,3X2E13.5,3X7HNODE 4,3X2E13.5) OUTP 790
      WRITE(6,1199) NFPCRL                      OUTP 800
1199 FORMAT( 1H0,77X 42HNUMBER OF UNCRUSHED FOOTPAD CRUSH LEVELS =,I2 /OUTP 810
123X 21HTOTAL ENERGY ABSORBED,38X 36HFOOTPAD ATTENUATION ENERGY THIOUTP 820
15  STEP /                                     OUTP 830
21X, 2(4X25HSURFACE COORDINATE SYSTEM9X24HLANDER COORDINATE SYSTEM OUTP 840
3 ,4X) / 1X4(5X1HX,10X1HY,10X1HZ,5X) )        OUTP 850
      WRITE(6,1200)(EGYXYZ(I),I=1,3),(EGYCDL(I),I=1,3), OUTP 860
1      (CGYXYZ(I),I=1,3),(CGYCDL(I),I=1,3)    OUTP 870
1200 FORMAT ( 1H0, 12E11.3 )                   OUTP 880
      RETURN                                    OUTP 890
      END                                        OUTP 900

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SUBROUTINE ENERGY( FORC2 )                                ENGY 10
DIMENSION FORC2(3)                                        ENGY 20
DIMENSION FORCL1(3), FORCL2(3), XYZDSL(3)                ENGY 30
DIMENSION FORC1(3), IXYZ(3)                             ENGY 40
COMMON / CMAIN /                                         ENGY 50
1  LGRTYP, LDIND , ITER, NPOINT, NREFP , NELEM , NUMB ,  ENGY 60
2  NSTEP , NITER , TOL , ENGABS, IFPATN, NFPCRL, FPCRLD(3), ENGY 70
3  IGLDIR, NMATL , IST , IMATL(4) , MNO , FPCRST(3),    ENGY 80
4  INDPL(4) , ELASBN(4) , ELASAX(4) ,                 ENGY 90
5  BINERT(4) , AREA(4) , IRET(4) ,                   ENGY 100
6  IFIRST(4) , SCMAX(4) , STMAX(4) ,                 ENGY 110
7  SRULC(4) , SRULT(4), CDC(5,4), SRC(5,4) ,          ENGY 120
8  CDT(5,4) , CRC(5,4) , SRT(5,4) ,                  ENGY 130
9  PFC(5,4) , PFT(5,4) , NPN , XYZPOS(3,5)           ENGY 140
9  , DC(3,3) , PHI , THTA , PSI                       ENGY 150
9  , CGYCDL(3), EGYCDL(3)                              ENGY 160
COMMON / CMAIN /
A  NITERS, NDISPS, STLN2 ,                              ENGY 170
B  IDSPDR, IDSPD1, IDSPD2, SPDISP, COEFF , XYZDSP(3),   ENGY 180
C  STLN(4) , STLNO(4) , STRP(4) ,                     ENGY 190
D  SPNGC(4) , SPNGT(4) , DISTC(4) ,                   ENGY 200
E  DISTT(4) , PFORC(4) , PFORT(4) ,                   ENGY 210
F  FOREVC(4) , FOREVT(4) , IPOSC(4) ,                  ENGY 220
G  IPOST(4) , INDULC(4) , INDULT(4) ,                  ENGY 230
H  STR(4) , AXIALF(4) , AXSTIF(4) ,                   ENGY 240
I  IPREV(4) , XYZDS(3) , SFCRCE(3) ,                   ENGY 250
J  CFRCE(30) , SUMFM(6) , R24XYZ(3) ,                  ENGY 260
K  TM(36) , FMVCTL(12,4) , FMVCTG(12,4) ,              ENGY 270
L  YSHRFP(4), ZSHRFP(4), YSHRMP(4), ZSHRMP(4), CGYXYZ(3), ENGY 280
M  YSHRFQ(4), ZSHRFQ(4), YSHRMQ(4), ZSHRMQ(4), EGYXYZ(3) ENGY 290
DATA IXYZ / 1HX, 1HY, 1HZ /                             ENGY 300
ZERO OUT FOOTPAD CRUSH ENERGY                           ENGY 310
DO 10 I = 1, 3                                           ENGY 320
10 CGYXYZ(I) = 0.0                                         ENGY 330
IF ( IFPATN .EQ. 2 ) GO TO 50                             ENGY 340
IF ( NFPCRL .EQ. 0 ) GO TO 50                             ENGY 350
C  CALCULATE THE FORCE COMPONENT AT THE FOOTPAD JOINT    ENGY 360
C  NORMAL TO THE LANDING SURFACE                          ENGY 370
C  RESULT = FORC2( IDSPDR )                               ENGY 380
C  ENGY 390
C  ENGY 400
C  CALCULATE THE FOOTPAD CRUSH ENERGY FOR THIS STEP IN SURFACE ENGY 410
C  COORDINATES AND ADD IT TO TOTAL ENERGY IN SURFACE   ENGY 412
C  COORDINATES                                           ENGY 414
C  ENGY 420
C  K = NFPCRL1                                           ENGY 430
C  J = NFPCRL                                             ENGY 440
C  DO 30 I = 1,J                                         ENGY 450
C  IF ( RESULT .LT. FPCRLD(K) ) GO TO 30                 ENGY 460
C  NFPCRL = NFPCRL - 1                                    ENGY 470
C  CGYXYZ(IDSPDR) = CGYXYZ(IDSPDR) + FPCRLD(K)*FPCRST(K) ENGY 480
30 K = K - 1                                             ENGY 490
C  IF ( NFPCRL .EQ. J ) GO TO 50                         ENGY 500
C  EGYXYZ(IDSPDR) = EGYXYZ(IDSPDR) + CGYXYZ(IDSPDR)     ENGY 510
50 CONTINUE                                             ENGY 520
C  ENGY 530

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C          CALCULATE FOOTPAD CRUSH ENERGY FOR THIS STEP IN LANDER          ENGY 540
C          COORDINATES BY DIRECT TRANSFORMATION.                            ENGY 550
C          CALCULATE ENERGY FOR THIS STEP DUE TO DISPLACEMENT OF FOOTPAD ENGY 560
C          JOINT IN BOTH SURFACE AND LANDER COORDINATES                     ENGY 565
C                                                                              ENGY 570
C          DO 410 I = 1, 3                                                  ENGY 580
C          CGYCDL(I) = 0.0                                                  ENGY 590
C          FORCL2(I) = 0.0                                                  ENGY 600
C          XYZDSL(I) = 0.0                                                  ENGY 610
C          DO 400 J = 1, 3                                                  ENGY 620
C          CGYCDL(I) = CGYCDL(I) + DC(J,I)*CGYXYZ(J)                       ENGY 630
C          FORCL2(I) = FORCL2(I) + DC(J,I)* FORC2(J)                       ENGY 640
C          XYZDSL(I) = XYZDSL(I) + DC(J,I)* XYZDS (J)                       ENGY 650
400 CONTINUE                                                                ENGY 660
C                                                                              ENGY 661
C          CALCULATE TOTAL ENERGY IN LANDER COORDINATES AS THE SUM OF     ENGY 662
C          THE PREVIOUS TOTAL AND THE FOOTPAD CRUSH ENERGY FOR            ENGY 663
C          THIS STEP AND THE ENERGY DUE TO FOOTPAD JOINT                  ENGY 664
C          DISPLACEMENT FOR THIS STEP                                      ENGY 665
C                                                                              ENGY 666
C          EGYCDL(I) = EGYCDL(I) + XYZDSL(I) *(FORCL1(I)+FORCL2(I)) *.5    ENGY 670
C          EGYCDL(I) = EGYCDL(I) + CGYCDL(I)                                ENGY 680
410 FORCL1(I) = FORCL2(I)                                                  ENGY 690
C                                                                              ENGY 700
C          CALCULATE TOTAL ENERGY IN SURFACE COORDINATES AS THE SUM OF    ENGY 710
C          THE PREVIOUS TOTAL AND THE FOOTPAD CRUSH ENERGY FOR            ENGY 720
C          THIS STEP AND THE ENERGY DUE TO FOOTPAD JOINT                  ENGY 722
C          DISPLACEMENT FOR THIS STEP                                      ENGY 724
C                                                                              ENGY 730
C          DO 60 I = 1, 3                                                  ENGY 740
C          EGYXYZ(I) = EGYXYZ(I) + XYZDS(I)*(FORC1(I) + FORC2(I)) *.5      ENGY 750
60 FORCL1(I) = FORC2(I)                                                  ENGY 760
C          CHECK ENERGY LIMIT                                            ENGY 770
C          IF ( IGLDIR .EQ. 0 ) GO TO 80                                    ENGY 780
C          IF ( ENGABS .GE. 0.) GO TO 64                                    ENGY 790
C          IF ( EGYXYZ(IGLDIR) .GT. ENGABS ) GO TO 80                      ENGY 800
C          GO TO 66                                                        ENGY 810
64 CONTINUE                                                                ENGY 820
C          IF ( EGYXYZ( IGLDIR ) .LT. ENGABS ) GO TO 80                    ENGY 830
66 CONTINUE                                                                ENGY 840
C          WRITE(6,71)                                                    ENGY 842
71 FORMAT(1H0, 33X 6(10H*****)) )                                       ENGY 844
C          WRITE(6,70)EGYXYZ(IGLDIR),IXYZ(IGLDIR),ENGABS                  ENGY 850
70 FORMAT(24H0**** THE TOTAL ENERGY (E14.7,                             ENGY 860
1 9H) IN THE A1,19H SURFACE COORDINATE,                                  ENGY 870
2 38H DIRECTION EXCEEDS THE ENERGY CUTOFF (,                             ENGY 880
3 E14.7,6H) ****)                                                        ENGY 885
C          IGLDIR = 0                                                    ENGY 890
80 CONTINUE                                                                ENGY 900
C          RETURN                                                         ENGY 910
C          ENTRY ENRG1                                                    ENGY 920
C          DO 100 I = 1, 3                                                ENGY 930
C          FORCL1(I) = 0.0                                                ENGY 940
C          FORC1(I) = 0.0                                                ENGY 950
100 EGYXYZ(I)= 0.0                                                        ENGY 960
C          NFPCR1 = NFPCR1                                               ENGY 970

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RETURN
END

ENGY 980
ENGY 990

	SUBROUTINE TRNFSM(STIFML, TRANSM,NORDER)	TRSM 10
	DIMENSION STIFML(NORDER,NORDER),TRANSM(3,3)	TRSM 20
	DIMENSION A(12), IJ(4), IJC(12)	TRSM 30
	DIMENSION IRC(12)	TRSM 40
	DATA IJ / 0,3,6,9 /	TRSM 50
	DATA IJC / 3*0, 3*3, 3*6, 3*9 /	TRSM 60
	DATA IRC / 1,2,3,1,2,3,1,2,3,1,2,3 /	TRSM 70
	ITINJO=1	TRSM 80
C		TRSM 90
C	MULTIPLY TRANSFORMATION MATRIX TIMES STIFFNESS MATRIX	TRSM 100
C	STORE RESULT IN STIFFNESS MATRIX.	TRSM 110
C		TRSM 120
	NORDR3 = NORDER / 3	TRSM 130
	DO 40 K = 1, NORDER	TRSM 140
	ICR=0	TRSM 150
	DO 30 IC =1, NORDR3	TRSM 160
	DO 30 I=1,3	TRSM 170
	ICR=ICR+1	TRSM 180
	A(ICR)=0.0	TRSM 190
	DO 30 J=1,3	TRSM 200
	JJ=J+IJ(IC)	TRSM 210
	IF (STIFML(K,JJ).EQ.0.) GO TO 30	TRSM 220
	A(ICR)=A(ICR)+STIFML(K,JJ)*TRANSM(J,I)	TRSM 230
30	CONTINUE	TRSM 240
	DO 40 J = 1, NORDER	TRSM 250
40	STIFML(K,J)=A(J)	TRSM 260
C		TRSM 270
C		TRSM 280
C	MULTIPLY THE RESULT OF THE ABOVE TIMES THE TRANSPOSE OF THE	TRSM 290
C	TRANSFORMATION MATRIX	TRSM 300
C		TRSM 310
	DO 60 K = 1, NORDER	TRSM 320
	DO 50 ICR = K, NORDER	TRSM 330
	A(ICR)=0.0	TRSM 340
	I=IRC(ICR)	TRSM 350
	DO 50 J=1,3	TRSM 360
	JJ=J+IJC(ICR)	TRSM 370
50	A(ICR)=A(ICR)+STIFML(JJ,K)*TRANSM(J,I)	TRSM 380
	DO 60 J = K, NORDER	TRSM 390
60	STIFML(J,K)=A(J)	TRSM 400
	ICR = NORDER - 1	TRSM 410
	DO 70 I = 1, ICR	TRSM 420
	IC=I+1	TRSM 430
	DO 70 J = IC,NORDER	TRSM 440
70	STIFML(I,J)=STIFML(J,I)	TRSM 450
C		TRSM 460
C	AS A RESULT OF THE ABOVE, STIFML CONTAINS THE STIFFNESS	TRSM 470
C	MATRIX TRANSFORMED TO THE GLOBAL COORDINATE SYSTEM	TRSM 480
C		TRSM 490
	RETURN	TRSM 500
	END	TRSM 510

	SUBROUTINE TRALMG(FMG, FML, TM, NG)	TRFM 10
C	F.M. GLOBAL = (TM)T . (F.M. LOCAL)	TRFM 20
C	FM. VECTOR =FXP,FYP,FZP,FXQ,FYQ,FZQ,MXP....,MXQ,MYQ,MZQ	TRFM 30
	DIMENSION FMG(12), FML(12), TM(3,3)	TRFM 40
	N = 0	TRFM 50
	M = 0	TRFM 60
	DO 30 I = 1, NG	TRFM 70
C		TRFM 80
C	TRANSFORM LOCAL FORCE-MOMENTS TO GLOBAL FORCE-MOMENTS	TRFM 90
C		TRFM 100
	DO 20 J = 1, 3	TRFM 110
	M = M + 1	TRFM 120
	FMG(M) = 0.0	TRFM 130
	DO 20 K = 1, 3	TRFM 140
	NK = N + K	TRFM 150
	FMG(M) = FMG(M) + TM(K,J) * FML(NK)	TRFM 160
	20 CONTINUE	TRFM 170
	30 N = N + 3	TRFM 180
	RETURN	TRFM 190
	END	TRFM 200

	SUBROUTINE STFMIT(STIFML, PO, DPQ, AE)	STAX 10
	DIMENSION STIFML(6,6)	STAX 20
C		STAX 30
C	COMPUTE THE LOCAL STIFFNESS MATRIX OF AN AXIAL STRUT	STAX 40
C		STAX 50
C	PO = CURRENT AXIAL FORCE	STAX 60
C	AE = CURRENT AXIAL STIFFNESS	STAX 70
C	DPQ= LENGTH OF THE PQ ELEMENT	STAX 80
	DO 100 I = 2, 6	STAX 90
	K = I - 1	STAX 100
	DO 100 J = 1, K	STAX 110
100	STIFML(I,J) = 0.00	STAX 120
	STIFML(1,1) = AE	STAX 130
	STIFML(2,2) = PO / DPQ	STAX 140
	STIFML(3,3) = STIFML(2,2)	STAX 150
	STIFML(4,4) = AE	STAX 160
	STIFML(5,5) = STIFML(2,2)	STAX 170
	STIFML(6,6) = STIFML(2,2)	STAX 180
	STIFML(4,1) = -AE	STAX 190
	STIFML(5,2) = -STIFML(2,2)	STAX 200
	STIFML(6,3) = -STIFML(2,2)	STAX 210
	DO 200 I = 2, 6	STAX 220
	K = I - 1	STAX 230
	DO 200 J = 1, K	STAX 240
200	STIFML(J,I) = STIFML(I,J)	STAX 250
	RETURN	STAX 260
	END	STAX 270

	SUBROUTINE TRANSN(P, Q, R, TM, TOTLPG)	TRAN 10
	DIMENSION P(3), Q(3), TM(3,3), R(3)	TRAN 20
	DOUBLE PRECISION X, Y, Z, TCTL, TMD(3,3), DPE, DER	TRAN 30
C	L1 M1 N1 THE TRANSFORMATION MATRIX REQUIRED TO	TRAN 40
C	TM = L2 M2 N2 CONVERT GLOBAL COORDINATES TO LOCAL	TRAN 50
C	L3 M3 N3 COORDINATES	TRAN 60
	X = Q(1) - P(1)	TRAN 70
	Y = Q(2) - P(2)	TRAN 80
	Z = Q(3) - P(3)	TRAN 90
C	COMPUTE THE TOTAL LENGTH OF THE BAR PQ	TRAN 100
	TOTL = DSQRT(X*X + Y*Y + Z*Z)	TRAN 110
	TOTLPG=TOTL	TRAN 120
C	COMPUTE THE DIRECTION COSINES OF THE LOCAL X AXIS PQ	TRAN 130
	TMD(1,1)= X / TOTL	TRAN 140
	TMD(1,2) = Y / TOTL	TRAN 150
	TMD(1,3) = Z / TOTL	TRAN 160
	X = R(1) - P(1)	TRAN 170
	Y = R(2) - P(2)	TRAN 180
	Z = R(3) - P(3)	TRAN 190
	DPE = TMD(1,1)*X + TMD(1,2)*Y + TMD(1,3)*Z	TRAN 200
	DER = DSQRT(X*X + Y*Y + Z*Z - DPE*DPE)	TRAN 210
C	COMPUTE THE DIRECTION COSINES OF THE LOCAL Y AXIS ER	TRAN 220
C	WHERE E IS THE POINT OF INTERSECTION ON PQ OF THE	TRAN 230
C	PERPENDICULAR DRAWN FROM R TO PQ	TRAN 240
	TMD(2,1) = (X - TMD(1,1)*DPE)/DER	TRAN 250
	TMD(2,2) = (Y - TMD(1,2)*DPE)/DER	TRAN 260
	TMD(2,3) = (Z - TMD(1,3) *DPE)/DER	TRAN 270
C	COMPUTE THE DIRECTION COSINES OF THE LOCAL Z AXIS FRXPQ	TRAN 280
	TMD(3,1) = TMD(1,2)*TMD(2,3) - TMD(2,2)*TMD(1,3)	TRAN 290
	TMD(3,2) = TMD(2,1)*TMD(1,3) - TMD(1,1)*TMD(2,3)	TRAN 300
	TMD(3,3) = TMD(1,1)*TMD(2,2) - TMD(2,1)*TMD(1,2)	TRAN 310
	DO 10 I=1,3	TRAN 320
	DO 10 J=1,3	TRAN 330
10	TM(I,J) = TMD(I,J)	TRAN 340
	RETURN	TRAN 350
	END	TRAN 360

	SUBROUTINE DMFSS(A,N,EPS,IRANK,TRAC)	DMFS 10
C		DMFS 20
C	SUBROUTINE DMFSS DETERMINES THE RANK (IRANK) AND LINEARLY	DMFS 30
C	INDEPENDENT ROWS AND COLUMNS OF A SYMMETRIC POSITIVE	DMFS 40
C	SEMI-DEFINITE MATRIX (A) OF ORDER N AND PREPARES THE MATRIX	DMFS 50
C	FOR CALCULATION OF THE LEAST SQUARES SOLUTION OF MINIMAL LENGTH	DMFS 60
C		DMFS 70
C		DMFS 80
C	DIMENSIONED DUMMY VARIABLES	DMFS 90
C	DIMENSION A(1),TRAC(1)	DMFS 100
C	DOUBLE PRECISION SUM,A,TRAC,PIV,HOLD	DMFS 110
C		DMFS 120
C	TEST OF SPECIFIED DIMENSION	DMFS 130
C	IF(N)36,36,1	DMFS 140
C		DMFS 150
C	INITIALIZE TRIANGULAR FACTORIZATION	DMFS 160
C	1 IRANK=0	DMFS 170
C	ISUB=0	DMFS 180
C	KPIV=0	DMFS 190
C	J=0	DMFS 200
C	PIV=0.D0	DMFS 210
C		DMFS 220
C	SEARCH FIRST PIVOT ELEMENT	DMFS 230
C	DO 3 K=1,N	DMFS 240
C	J=J+K	DMFS 250
C	TRAC(K)=A(J)	DMFS 260
C	IF(A(J)-PIV)3,3,2	DMFS 270
C	2 PIV=A(J)	DMFS 280
C	KSUB=J	DMFS 290
C	KPIV=K	DMFS 300
C	3 CONTINUE	DMFS 310
C		DMFS 320
C	START LOOP OVER ALL ROWS OF A	DMFS 330
C	DO 32 I=1,N	DMFS 340
C	ISUB=ISUB+I	DMFS 350
C	IM1=I-1	DMFS 360
C	4 KMI=KPIV-I	DMFS 370
C	IF(KMI)35,9,5	DMFS 380
C		DMFS 390
C	PERFORM PARTIAL COLUMN INTERCHANGE	DMFS 400
C	5 JI=KSUB-KMI	DMFS 410
C	IDC=JI-ISUB	DMFS 420
C	JJ=ISUB-IM1	DMFS 430
C	DO 6 K=JJ,ISUB	DMFS 440
C	KK=K+IDC	DMFS 450
C	HOLD=A(K)	DMFS 460
C	A(K)=A(KK)	DMFS 470
C	6 A(KK)=HOLD	DMFS 480
C		DMFS 490
C	PERFORM PARTIAL ROW INTERCHANGE	DMFS 500
C	KK=KSUB	DMFS 510
C	DO 7 K=KPIV,N	DMFS 520
C	II=KK-KMI	DMFS 530
C	HOLD=A(KK)	DMFS 540
C	A(KK)=A(II)	DMFS 550

	A(II)=HOLD	DMFS 560
	7 KK=KK+K	DMFS 570
C		DMFS 580
C	PERFORM REMAINING INTERCHANGE	DMFS 590
	JJ=KPIV-1	DMFS 600
	II=ISUB	DMFS 610
	DO 8 K=I,JJ	DMFS 620
	HOLD=A(II)	DMFS 630
	A(II)=A(JI)	DMFS 640
	A(JI)=HOLD	DMFS 650
	II=II+K	DMFS 660
	8 JI=JI+1	DMFS 670
	9 IF(IRANK)22,10,10	DMFS 680
C		DMFS 690
C	RECORD INTERCHANGE IN TRANSPOSITION VECTOR	DMFS 700
	10 TRAC(KPIV)=TRAC(I)	DMFS 710
	TRAC(I)=KPIV	DMFS 720
C		DMFS 730
C	MODIFY CURRENT PIVOT ROW	DMFS 740
	KK=IM1-IRANK	DMFS 750
	KMI=ISUB-KK	DMFS 760
	PIV=0.00	DMFS 770
	IDC=IRANK+1	DMFS 780
	JI=ISUB-1	DMFS 790
	JK=KMI	DMFS 800
	JJ=ISUB-I	DMFS 810
	DO 19 K=I,N	DMFS 820
	SUM=0.00	DMFS 830
C		DMFS 840
C	BUILD UP SCALAR PRODUCT IF NECESSARY	DMFS 850
	IF(KK)13,13,11	DMFS 860
	11 DO 12 J=KMI,JI	DMFS 870
	SUM=SUM-A(J)*A(JK)	DMFS 880
	12 JK=JK+1	DMFS 890
	13 JJ=JJ+K	DMFS 900
	IF(K-I)14,14,16	DMFS 910
	14 SUM=A(ISUB)+SUM	DMFS 920
C		DMFS 930
C	TEST RADICAND FOR LOSS OF SIGNIFICANCE	DMFS 940
	IF(SUM-DABS(A(ISUB)*DBLE(EPS)))20,20,15	DMFS 950
	15 A(ISUB)=DSQRT(SUM)	DMFS 960
	KPIV=I+1	DMFS 970
	GOTO 19	DMFS 980
	16 SUM=(A(JK)+SUM)/A(ISUB)	DMFS 990
	A(JK)=SUM	DMFS1000
C		DMFS1010
C	SEARCH FOR NEXT PIVOT ROW	DMFS1020
	IF(A(JJ))19,19,17	DMFS1030
	17 TRAC(K)=TRAC(K)-SUM*SUM	DMFS1040
	HOLD=TRAC(K)/A(JJ)	DMFS1050
	IF(PIV-HOLD)18,19,19	DMFS1060
	18 PIV=HOLD	DMFS1070
	KPIV=K	DMFS1080
	KSUB=JJ	DMFS1090
	19 JK=JJ+IDC	DMFS1100
	GOTO 32	DMFS1110

C		DMFS1120
C	CALCULATE MATRIX OF DEPENDENCIES U	DMFS1130
	20 IF (IRANK)21,21,37	DMFS1140
	21 IRANK=-1	DMFS1150
	GOTO 4	DMFS1160
	22 IRANK=IM1	DMFS1170
	II=ISUB-IRANK	DMFS1180
	JI=II	DMFS1190
	DO 26 K=1,IRANK	DMFS1200
	JI=JI-1	DMFS1210
	JK=ISUB-1	DMFS1220
	JJ=K-1	DMFS1230
	DO 26 J=I,N	DMFS1240
	IDC=IRANK	DMFS1250
	SUM=0.D0	DMFS1260
	KMI=JI	DMFS1270
	KK=JK	DMFS1280
	IF (JJ)25,25,23	DMFS1290
	23 DO 24 L=1,JJ	DMFS1300
	IDC=IDC-1	DMFS1310
	SUM=SUM-A(KMI)*A(KK)	DMFS1320
	KMI=KMI-IDC	DMFS1330
	24 KK=KK-1	DMFS1340
	25 A(KK)=(SUM+A(KK))/A(KMI)	DMFS1350
	26 JK=JK+J	DMFS1360
C		DMFS1370
C	CALCULATE I+TRANSPOSE(U)*U	DMFS1380
	JJ=ISUB-I	DMFS1390
	PIV=0.D0	DMFS1400
	KK=ISUB-1	DMFS1410
	DO 31 K=I,N	DMFS1420
	JJ=JJ+K	DMFS1430
	IDC=0	DMFS1440
	DO 28 J=K,N	DMFS1450
	SUM=0.D0	DMFS1460
	KMI=JJ+IDC	DMFS1470
	DO 27 L=II,KK	DMFS1480
	JK=L+IDC	DMFS1490
	27 SUM=SUM+A(L)*A(JK)	DMFS1500
	A(KMI)=SUM	DMFS1510
	28 IDC=IDC+J	DMFS1520
	A(JJ)=A(JJ)+1.D0	DMFS1530
	TRAC(K)=A(JJ)	DMFS1540
C		DMFS1550
C	SEARCH NEXT DIAGONAL ELEMENT	DMFS1560
	IF (PIV-A(JJ))29,30,30	DMFS1570
	29 KPIV=K	DMFS1580
	KSUB=JJ	DMFS1590
	PIV=A(JJ)	DMFS1600
	30 II=II+K	DMFS1610
	KK=KK+K	DMFS1620
	31 CONTINUE	DMFS1630
	GOTO 4	DMFS1640
	32 CONTINUE	DMFS1650
	33 IF (IRANK)35,34,35	DMFS1660
	34 IRANK=N	DMFS1670

35	RETURN	DMFS1680
C		DMFS1690
C	ERROR RETURNS	DMFS1700
C		DMFS1710
C	RETURN IN CASE OF ILLEGAL DIMENSION	DMFS1720
36	IRANK=-1	DMFS1730
	RETURN	DMFS1740
C		DMFS1750
C	INSTABLE FACTORIZATION OF I+TRANSPPOSE(U)*U	DMFS1760
37	IRANK=-2	DMFS1770
	RETURN	DMFS1780
	END	DMFS1790

	SUBROUTINE DMLSS(A,N,IRANK,TRAC,INC,RHS,IER)	DMLS 10
C		DMLS 20
C	SUBROUTINE DMLSS CALCULATES THE LEAST SQUARES SOLUTION OF	DMLS 30
C	MINIMAL LENGTH OF A SYSTEM OF SIMULTANEOUS LINEAR EQUATIONS	DMLS 40
C	WITH SYMMETRIC POSITIVE SEMI-DEFINITE COEFFICIENT MATRIX (A)	DMLS 50
C	WHOSE RANK (IRANK) IS KNOWN AND MAY BE LESS THAN THE ORDER (N)	DMLS 60
C		DMLS 70
C		DMLS 80
C	DIMENSIONED DUMMY VARIABLES	DMLS 90
C	DIMENSION A(1),TRAC(1),RHS(1)	DMLS 100
C	DOUBLE PRECISION SUM,A,RHS,TRAC,HOLD	DMLS 110
C		DMLS 120
C	TEST OF SPECIFIED DIMENSIONS	DMLS 130
C	IDEF=N-IRANK	DMLS 140
C	IF(N)33,33,1	DMLS 150
C	1 IF(IRANK)33,33,2	DMLS 160
C	2 IF(IDEF)33,3,3	DMLS 170
C		DMLS 180
C	CALCULATE AUXILIARY VALUES	DMLS 190
C	3 ITE=IRANK*(IRANK+1)/2	DMLS 200
C	IX2=IRANK+1	DMLS 210
C	NP1=N+1	DMLS 220
C	IFR=0	DMLS 230
C		DMLS 240
C	INTERCHANGE RIGHT HAND SIDE	DMLS 250
C	JJ=1	DMLS 260
C	II=1	DMLS 270
C	4 DO 6 I=1,N	DMLS 280
C	J=TRAC(II)	DMLS 290
C	IF(J)31,31,5	DMLS 300
C	5 HOLD=RHS(II)	DMLS 310
C	RHS(II)=RHS(J)	DMLS 320
C	RHS(J)=HOLD	DMLS 330
C	6 II=II+JJ	DMLS 340
C	IF(JJ)32,7,7	DMLS 350
C		DMLS 360
C	PERFORM STEP 2 IF NECESSARY	DMLS 370
C	7 ISW=1	DMLS 380
C	IF(INC*IDEF)8,28,8	DMLS 390
C		DMLS 400
C	CALCULATE X1 = X1 + U * X2	DMLS 410
C	8 ISTA=ITE	DMLS 420
C	DO 10 I=1,IRANK	DMLS 430
C	ISTA=ISTA+1	DMLS 440
C	JJ=ISTA	DMLS 450
C	SUM=0.D0	DMLS 460
C	DO 9 J=IX2,N	DMLS 470
C	SUM=SUM+A(JJ)*RHS(J)	DMLS 480
C	9 JJ=JJ+J	DMLS 490
C	10 RHS(I)=RHS(I)+SUM	DMLS 500
C	GOTO(11,28,11),ISW	DMLS 510
C		DMLS 520
C	CALCULATE X2 = TRANSPOSE(U) * X1	DMLS 530
C	11 ISTA=ITE	DMLS 540
C	DO 15 I=IX2,N	DMLS 550

	JJ=ISTA	DMLS 560
	SUM=0.D0	DMLS 570
	DO 12 J=1,IRANK	DMLS 580
	JJ=JJ+1	DMLS 590
12	SUM=SUM+A(JJ)*RHS(J)	DMLS 600
	GOTO(13,13,14),ISW	DMLS 610
13	SUM=-SUM	DMLS 620
14	RHS(I)=SUM	DMLS 630
15	ISTA=ISTA+I	DMLS 640
	GOTO(16,29,30),ISW	DMLS 650
C		DMLS 660
C	INITIALIZE STEP (4) OR STEP (8)	DMLS 670
16	ISTA=IX2	DMLS 680
	IEND=N	DMLS 690
	JJ=ITE+ISTA	DMLS 700
C		DMLS 710
C	DIVISION OF X1 BY TRANSPOSE OF TRIANGULAR MATRIX	DMLS 720
17	SUM=0.D0	DMLS 730
	DO 20 I=ISTA,IEND	DMLS 740
	IF(A(JJ))18,31,18	DMLS 750
18	RHS(I)=(RHS(I)-SUM)/A(JJ)	DMLS 760
	IF(I-IEND)19,21,21	DMLS 770
19	JJ=JJ+ISTA	DMLS 780
	SUM=0.D0	DMLS 790
	DO 20 J=ISTA,I	DMLS 800
	SUM=SUM+A(JJ)*RHS(J)	DMLS 810
20	JJ=JJ+1	DMLS 820
C		DMLS 830
C	DIVISION OF X1 BY TRIANGULAR MATRIX	DMLS 840
21	SUM=0.D0	DMLS 850
	II=IEND	DMLS 860
	DO 24 I=ISTA,IEND	DMLS 870
	RHS(II)=(RHS(II)-SUM)/A(JJ)	DMLS 880
	IF(II-ISTA)25,25,22	DMLS 890
22	KK=JJ-1	DMLS 900
	SUM=0.D0	DMLS 910
	DO 23 J=II,IEND	DMLS 920
	SUM=SUM+A(KK)*RHS(J)	DMLS 930
23	KK=KK+J	DMLS 940
	JJ=JJ-II	DMLS 950
24	II=II-1	DMLS 960
25	IF(IDEF)26,30,26	DMLS 970
26	GOTO(27,11,8),ISW	DMLS 980
C		DMLS 990
C	PERFORM STEP (5)	DMLS1000
27	ISW=2	DMLS1010
	GOTO 8	DMLS1020
C		DMLS1030
C	PERFORM STEP (6)	DMLS1040
28	ISTA=1	DMLS1050
	IEND=IRANK	DMLS1060
	JJ=1	DMLS1070
	ISW=2	DMLS1080
	GOTO 17	DMLS1090
C		DMLS1100
C	PERFORM STEP (8)	DMLS1110

29	ISW=3	DMLS1120
	GOTO 16	DMLS1130
C		DMLS1140
C	REINTERCHANGE CALCULATED SOLUTION	DMLS1150
30	II=N	DMLS1160
	JJ=-1	DMLS1170
	GOTO 4	DMLS1180
C		DMLS1190
C	ERROR RETURN IN CASE OF ZERO DIVISOR	DMLS1200
31	IER=1	DMLS1210
32	RETURN	DMLS1220
C		DMLS1230
C	ERROR RETURN IN CASE OF ILLEGAL DIMENSION	DMLS1240
33	IER=-1	DMLS1250
	RETURN	DMLS1260
	END	DMLS1270

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SUBROUTINE STRUT ( IST , SAVE1 , SAVE2 , SAVE3 , SAVE4 ,          STRT  10
1          SAVE5 , SAVE6 , SAVE7 , SAVE8 , ISAVE1 ,          STRT  20
2          ISAVE2 , ISAVE3 , STRP , IPOSC , IPOST ,          STRT  30
3          IRET , SCMAX , STMAX , CDC , SRULT ,              STRT  40
4          SRULC , SRT , SRC , PFT , PFC ,                  STRT  50
5          STR , FORCE , AXSTIF , IFIRST , ITER ,            STRT  60
6          CDT , ISTOP2 )                                    STRT  70
    DIMENSION CDC(1), CDT(1), SRT(1), SRC(1), PFT(1), PFC(1)  STRT  80
C
C THIS SUBROUTINE DETERMINES THE FORCE AND AXIAL STIFFNESS OF A STRT 100
C TYPICAL LANDING GEAR STRUT STRT 110
C STRT 120
C STRT 130
C INITIALIZE SUBROUTINE STRT 140
C STRT 150
    ISTOP2 = 0 STRT 155
    ABSTR=ABS(STR) STRT 160
    SPNGC=SAVE1 STRT 170
    SPNGT=SAVE2 STRT 180
    DISTC=SAVE3 STRT 190
    DISTT=SAVE4 STRT 200
    PFORC=SAVE5 STRT 210
    PFORT=SAVE6 STRT 220
    FOREVC=SAVE7 STRT 230
    FOREVT=SAVE8 STRT 240
    INDULC=ISAVE1 STRT 250
    INDULT=ISAVE2 STRT 260
    IJKC=1 STRT 270
    IJKT=1 STRT 280
    IPREV=ISAVE3 STRT 290
C STRT 300
C DETERMINE STROKING DIRECTION STRT 310
C STRT 320
    IF(STR.LT.0.0) GO TO 500 STRT 330
    IF(STR.GT.0.0)GO TO 400 STRT 340
C STRT 350
C * * * * * STRT 360
C ZERO STRUT STROKE STRT 370
C * * * * * STRT 380
C STRT 390
    FORCE=0.0 STRT 400
    GO TO 2002 STRT 410
C STRT 420
C * * * * * STRT 430
C STRUT IN TENSION STRT 440
C * * * * * STRT 450
C STRT 460
400 CONTINUE STRT 470
    IF(ABSTR.GT.STMAX.AND.ITER.EQ.1) GO TO 5001 STRT 480
C STRT 490
C WAS PREVIOUS STEP ON COMPRESSION SIDE STRT 500
C STRT 510
    IF(IPREV.EQ.0)GO TO 402 STRT 520
    IF(IPREV.NE.-1)GO TO 402 STRT 530
    IF(PFORC.EQ.0.0)GO TO 402 STRT 540

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	IPRFV=+1	STRT 550
	IF(IRET.EQ.1) GO TO 651	STRT 560
	GO TO 626	STRT 570
402	CONTINUE	STRT 580
	IPREV=+1	STRT 590
	IF(ABSTR.GT.DISTT)GO TO 401	STRT 600
	FORCE=0.0	STRT 610
	GO TO 2001	STRT 620
401	CONTINUE	STRT 630
	IF(STR.LT.STRP)GO TO 425	STRT 640
C		STRT 650
C	STRUT LOADING	STRT 660
C		STRT 670
	IF(SPNGT.EQ.0.0)GO TO 405	STRT 680
	FORCE=SPNGT*(ABSTR-DISTT)	STRT 690
	IF(FORCE.LT.FOREVT)GO TO 2001	STRT 700
	FOREVT=0.0	STRT 710
	INDULT=0	STRT 720
	SPNGT=0.0	STRT 730
	IF(IJKT.EQ.1)GO TO 407	STRT 740
	DISTT=CDT(IJKT-1)-PFT(IJKT-1)/SRT(IJKT)	STRT 750
	GO TO 408	STRT 760
407	DISTT=0.0	STRT 770
408	CONTINUE	STRT 780
	IF (FORCE.LT.PFT(IJKT))GO TO 405	STRT 790
	FORCE=PFT(IJKT)	STRT 800
	IF(ABSTR.LT.CDT(IJKT))GO TO 2001	STRT 810
405	CONTINUE	STRT 820
406	IF(ABSTR.LT.CDT(IJKT))GO TO 410	STRT 830
	DISTT=CDT(IJKT)-PFT(IJKT)/SRT(IJKT+1)	STRT 840
	IJKT=IJKT+1	STRT 850
	GO TO 406	STRT 860
410	CONTINUE	STRT 870
	FORCE=SRT(IJKT)*(ABSTR-DISTT)	STRT 880
	IF(FORCE.GT.PFT(IJKT))FORCE=PFT(IJKT)	STRT 890
	GO TO 2001	STRT 900
C		STRT 910
C	STRUT UNLOADING	STRT 920
C		STRT 930
425	CONTINUE	STRT 940
	IF(IRET.EQ.1)GO TO 450	STRT 950
	IF(INDULT.NE.0)GO TO 475	STRT 960
426	SPNGT=SRULT	STRT 970
	IF(PFORT.NE.PFT(IPOST))SPNGT=SRT(IPOST)	STRT 975
	FOREVT=PFORT	STRT 980
	GO TO 470	STRT 990
450	CONTINUE	STRT1000
	IF(INDULT.NE.0)GO TO 475	STRT1010
451	CONTINUE	STRT1020
	FOREVT=PFORT	STRT1030
	IF(PFORT.EQ.PFT(IPOST)) GO TO 465	STRT1040
	SPNGT=SRT(IPOST)	STRT1050
	GO TO 470	STRT1060
465	CONTINUE	STRT1070
	SPNGT=SRULT	STRT1080
	IF(IPOST.EQ.5) GO TO 470	STRT1090

	SPNGT=SRT(IPOST+1)	STRT1100
470	CONTINUE	STRT1110
	INDULT=1	STRT1120
	DISTT=ABS(STRP)-PFORT/SPNGT	STRT1130
	IF(IPREV.FQ.-1)GO TO 602	STRT1140
475	CONTINUE	STRT1150
	FORCE=SPNGT*(ABSTR-DISTT)	STRT1160
	IF(ABSTR.LT.DISTT)FORCE=0.0	STRT1170
	GO TO 2001	STRT1180
C		STRT1190
C	* * * * *	STRT1200
C	STRUT IN COMPRESSION	STRT1210
C	* * * * *	STRT1220
C		STRT1230
500	CONTINUE	STRT1240
	IF(ABSTR.GT.SCMAX.AND.ITER.EQ.1) GO TO 5000	STRT1250
C		STRT1260
C	WAS PREVIOUS STEP ON TENSION SIDE	STRT1270
C		STRT1280
	IF(IPREV.EQ.0)GO TO 602	STRT1290
	IF(IPREV.NE.+1)GO TO 602	STRT1300
	IF(PFORT.EQ.0.0)GO TO 602	STRT1310
	IPREV=-1	STRT1320
	IF(IRET.EQ.+1) GO TO 451	STRT1330
	GO TO 426	STRT1340
602	CONTINUE	STRT1350
	IPREV=-1	STRT1360
	IF(ABSTR.GT.DISTC)GO TO 600	STRT1370
	FORCE=0.0	STRT1380
	GO TO 2000	STRT1390
600	CONTINUE	STRT1400
	IF(ITER.NE.1.AND.PFORC.NE.PFC(IPOSC)) GO TO 699	STRT1410
	IF(STR.GT.STRP)GO TO 625	STRT1420
699	CONTINUE	STRT1430
C		STRT1440
C	STRUT LOADING	STRT1450
C		STRT1460
	IF(SPNGC.EQ.0.0)GO TO 605	STRT1470
	FORCE=SPNGC*(ABSTR-DISTC)	STRT1480
	IF(FORCE.LT.FOREVC)GO TO 2000	STRT1490
	FOREVC=0.0	STRT1500
	INDULC=0	STRT1510
	SPNGC=0.0	STRT1520
	IF(IJKC.EQ.1)GO TO 607	STRT1530
	DISTC=CDC(IJKC-1)-PFC(IJKC-1)/SRC(IJKC)	STRT1540
	GO TO 608	STRT1550
607	DISTC=0.0	STRT1560
608	CONTINUE	STRT1570
	IF(FORCE.LT.PFC(IJKC))GO TO 605	STRT1580
	FORCE=PFC(IJKC)	STRT1590
	IF(ABSTR.LT.CDC(IJKC))GO TO 2000	STRT1600
605	CONTINUE	STRT1610
606	IF(ABSTR.LT.CDC(IJKC))GO TO 610	STRT1620
	DISTC=CDC(IJKC)-PFC(IJKC)/SRC(IJKC+1)	STRT1630
	IJKC=IJKC+1	STRT1640
	GO TO 606	STRT1650

610	CONTINUE	STRT1660
	FORCE=SRC(IJKC)*(ABSTR-DISTC)	STRT1670
	IF(FORCE.GT.PFC(IJKC))FORCE=PFC(IJKC)	STRT1680
	GO TO 2000	STRT1690
C		STRT1700
C	STRUT UNLOADING	STRT1710
C		STRT1720
625	CONTINUE	STRT1730
	IF(IRET.EQ.1)GO TO 650	STRT1740
	IF(INDULC.NE.0)GO TO 675	STRT1750
626	SPNGC=SRULC	STRT1760
	IF(PFORC.NE.PFC(IPOSC))SPNGC=SRC(IPOSC)	STRT1765
	FOREVC=PFORC	STRT1770
	GO TO 670	STRT1780
650	CONTINUE	STRT1790
	IF(INDULC.NE.0)GO TO 675	STRT1800
651	CONTINUE	STRT1810
	FOREVC=PFORC	STRT1820
	IF(PFORC.EQ.PFC(IPOSC)) GO TO 665	STRT1830
	SPNGC=SRC(IPOSC)	STRT1840
	GO TO 670	STRT1850
665	CONTINUE	STRT1860
	SPNGC=SRULC	STRT1870
	IF(IPOSC.EQ.5) GO TO 670	STRT1880
	SPNGC=SRC(IPOSC+1)	STRT1890
670	CONTINUE	STRT1900
	INDULC=1	STRT1910
	DISTC=ABS(STRP)-PFORC/SPNGC	STRT1920
	IF(IPREV.EQ.+1)GO TO 402	STRT1930
675	CONTINUE	STRT1940
	FORCE=SPNGC*(ABSTR-DISTC)	STRT1950
	IF(ABSTR.LT.DISTC)FORCE=0.0	STRT1960
C		STRT1970
C	UPDATE INDICATORS	STRT1980
C		STRT1990
2000	CONTINUE	STRT2000
	PFORC=FORCE	STRT2010
	GO TO 2005	STRT2020
2001	CONTINUE	STRT2030
	PFORT=FORCE	STRT2040
	GO TO 2005	STRT2050
2002	CONTINUE	STRT2060
	PFORC=0.0	STRT2070
	PFORT=0.0	STRT2080
2005	CONTINUE	STRT2090
	IF(ITER.NE.1) GO TO 60	STRT2100
	IPOSC=IJKC	STRT2110
	IPOST=IJKT	STRT2120
	SAVE1=SPNGC	STRT2130
	SAVE2=SPNGT	STRT2140
	SAVE3=DISTC	STRT2150
	SAVE4=DISTT	STRT2160
	SAVE5=PFORC	STRT2170
	SAVE6=PFORT	STRT2180
	SAVE7=FOREVC	STRT2190
	SAVE8=FOREVT	STRT2200

ISAVE1=INDULC	STRT2210
ISAVE2=INDULT	STRT2220
STRP=STR	STRT2230
ISAVE3=IPREV	STRT2240
60 CONTINUE	STRT2250
IF(STR.EQ.0.)GO TO 100	STRT2260
FORCE = (STR/ABSTR)*FORCE	STRT2270
GO TO 101	STRT2280
100 FORCE=0.	STRT2290
101 CONTINUE	STRT2300
C	STRT2302
C CALCULATE AXIAL STIFFNESS	STRT2304
C	STRT2306
IF(STR.EQ.0.) GO TO 10	STRT2310
IF(STR.GT.0.) GO TO 15	STRT2320
IF(FORCE.EQ.0.)GO TO 12	STRT2330
IF(ABS(FORCE).EQ.PFC(IJKC)) GO TO 12	STRT2340
IF(SPNGC.EQ.0.)GO TO 14	STRT2350
AXSTIF=SPNGC	STRT2360
GO TO 50	STRT2370
14 AXSTIF=SRC(IJKC)	STRT2380
GO TO 50	STRT2390
15 IF(FORCE.EQ.0.)GO TO 12	STRT2400
11 IF(FORCE.EQ.PFT(IJKT)) GO TO 12	STRT2410
IF(SPNGT.EQ.0.)GO TO 13	STRT2420
AXSTIF=SPNGT	STRT2430
GO TO 50	STRT2440
13 AXSTIF=SRT(IJKT)	STRT2450
GO TO 50	STRT2460
12 AXSTIF=0.	STRT2470
GO TO 50	STRT2480
10 IF(IFIRST.LT.0) GO TO 20	STRT2490
AXSTIF=SRT(1)	STRT2500
GO TO 50	STRT2510
20 AXSTIF=SRC(1)	STRT2520
50 CONTINUE	STRT2530
RETURN	STRT2540
C	STRT2550
C STRUT BOTTOMED OUT ON COMPRESSION SIDE	STRT2560
C	STRT2570
5000 CONTINUE	STRT2580
WRITE(6,9000)IST	STRT2590
GO TO 5002	STRT2600
C	STRT2610
C STRUT BOTTOMED OUT ON TENSION SIDE	STRT2620
C	STRT2630
5001 CONTINUE	STRT2640
WRITE(6,9001)IST	STRT2650
5002 CONTINUE	STRT2660
ISTOP2 = -1	STRT2670
RETURN	STRT2680
C	STRT2690
C FORMAT STATEMENTS	STRT2700
C	STRT2710
9000 FORMAT(///,10X,*STRUT*,I3,* BOTTOMED OUT ON COMPRESSION SIDE*)	STRT2720
9001 FORMAT(///,10X,*STRUT*,I3,* BOTTOMED OUT ON TENSION SIDE*)	STRT2730

END

STRT2740

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PROGRAM INPUT
DIMENSION SAV(3)
DIMENSION ALPHA(8)
COMMON / CMAIN /
1 LGRTYP, LDIND , ITER, NPOINT, NREFP , NELEM , NUMB , INPT 10
2 NSTEP , NITER , TOL , ENGABS, IFPATN, NFPCRL, FPCRLD(3), INPT 20
3 IGLDIR, NMATL , IST , IMATL(4) , MNO , FPCRST(3), INPT 30
4 INDPL(4) , ELASBN(4) , ELASAX(4) , INPT 40
5 BINERT(4) , AREA(4) , IRET(4) , INPT 50
6 IFIRST(4) , SCMAX(4) , STMAX(4) , INPT 60
7 SRULC(4) , SRULT(4), CDC(5,4), SRC(5,4) , INPT 70
8 CDT(5,4) , CRC(5,4) , SRT(5,4) , INPT 80
9 PFC(5,4) , PFT(5,4) , NPN , XYZPOS(3,5) INPT 90
9 , DC(3,3) , PHI , THTA , PSI INPT 100
9 , CGYCDL(3), EGYCDL(3) INPT 110
COMMON / CMAIN / INPT 120
A NITERS, NDISPS, STLN2 , INPT 130
B IDSPDR, IDSPD1, IDSPD2, SPDISP, COEFF , XYZDSP(3), INPT 140
C STLN(4) , STLNO(4) , STRP(4) , INPT 150
D SPNGC(4) , SPNGT(4) , DISTC(4) , INPT 160
E DISTT(4) , PFORC (4) , PFORT(4) , INPT 170
F FOREVC(4) , FOREVT(4) , IPOSC(4) , INPT 180
G IPOST(4) , INDULC(4) , INDULT(4) , INPT 190
H STR(4) , AXIALF(4) , AXSTIF(4) , INPT 200
I IPREV(4) , XYZDS (3) , SFORCE(30) , INPT 210
J CFORCF(30) , SUMFM4(6) , R24XYZ(3) , INPT 220
K TM(36) , FMVCTL(12,4) , FMVCTG(12,4) , INPT 230
L YSHRFP(4), ZSHRFP(4), YSHRMP(4), ZSHRMP(4), CGYXYZ(3), INPT 240
M YSHRFQ(4), ZSHRFQ(4), YSHRMQ(4), ZSHRMQ(4), EGYXYZ(3) INPT 250
DATA DEGRAD / 57.295779513 / INPT 260
DATA BLANK / 1H / INPT 270
WRITE(6,40) INPT 280
40 FORMAT ( 37H GEAR ANALYSIS DATA - CARD CODE // INPT 290
1 28X31HBLANK - 0 COMMENT CARDS / INPT 300
2 28X44H 1 GEAR AND LOAD CARD / INPT 310
3 23X47H 2 FRICTION CARD / INPT 320
4 28X51H 3 APPLIED DISPLACEMENT CARD / INPT 330
5 28X45H 4 STRUT MATERIAL CARDS / INPT 340
6 28X49H 5 SOLUTION PARAMETER CARD / INPT 350
7 28X41H 6 NODAL POINT CARDS / INPT 360
8 28X38H 7 FOOTPAD CARD / INPT 370
9 28X49H 8 MATERIAL PARAMETER CARDS / INPT 380
A 28X44H 9 MATERIAL CRUSH CARDS / INPT 390
B 28X51H 10 COMPRESSION CRUSH DISTANCE CARDS /INPT 400
C 28X51H 11 TENSION CRUSH DISTANCE CARDS /INPT 410
D 28X51H 12 COMPRESSION SPRING RATE CARDS /INPT 420
E 28X51H 13 TENSION SPRING RATE CARDS /INPT 430
F 28X51H 14 COMPRESSION PLASTIC FORCE CARDS /INPT 440
G 28X51H 15 TENSION PLASTIC FORCE CARDS /INPT 450
H 28X42H 16 DATA TERMINATOR CARD / ) INPT 460
C INPT 470
C READ THE INPUT FILE AND PLACE ON UNIT 1 FOR CARD CODE INPT 480

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C		PROCESSING	INPT 530
	REWIND 1		INPT 540
	90 READ(5,100)(ALPHA(I),I=1,8)		INPT 550
	100 FORMAT (8A10)		INPT 560
	IF (FOF, 5) 103, 105		INPT 570
	103 CONTINUE		INPT 580
	WRITE(6,290)		INPT 590
	290 FORMAT(39H1END OF JOB---END OF DATA SET ON UNIT 5)		INPT 600
	STOP		INPT 610
	105 CONTINUE		INPT 620
	WRITE(6,1111)(ALPHA(I),I=1,8)		INPT 630
	DECODE (2, 451, ALPHA(1)) I		INPT 640
	451 FORMAT (I2)		INPT 650
	IF (I .EQ. 0) GO TO 90		INPT 660
	1111 FORMAT (1X8A10)		INPT 670
C	CHECK TO INSURE THAT CARD CODE IS RIGHT JUSTIFIED		INPT 680
	DECODE (2, 106, ALPHA(1)) A, B		INPT 690
	IF (B .NE. BLANK) GO TO 107		INPT 700
	WRITE(1,106) B, A		INPT 710
	106 FORMAT (2A1)		INPT 720
	GO TO 108		INPT 730
	107 CONTINUE		INPT 740
	WRITE(1,110) ALPHA(1)		INPT 750
	DECODE (2,280,ALPHA(1)) I		INPT 760
	IF (I .GT. 15) GO TO 120		INPT 770
	280 FORMAT (I2)		INPT 780
	110 FORMAT (A2)		INPT 790
	108 WRITE(1,100)(ALPHA(I),I=1,8)		INPT 800
	GO TO 90		INPT 810
	120 CONTINUE		INPT 820
	REWIND 1		INPT 830
C		END OF INITIAL READ	INPT 840
	150 READ(1,200) ICCODE		INPT 850
	200 FORMAT (I2)		INPT 860
	IF (EOF, 1) 1000, 205		INPT 870
	205 CONTINUE		INPT 880
C			INPT 890
C	ONLY CARDS WITH CARD CODES OF 1-15 ARE ACCEPTABLE		INPT 900
C			INPT 910
	IF (ICCODE .GT. 15) GO TO 1000		INPT 920
	IF (ICCODE .GT. 0) GO TO 220		INPT 930
	READ(1,100)(ALPHA(I),I=1,8)		INPT 940
	WRITE(6,210)(ALPHA(I),I=1,8)		INPT 950
	210 FORMAT (28H *****ERROR***** BAD CARD CODE,12X8A10)		INPT 960
	GO TO 150		INPT 970
	220 CONTINUE		INPT 980
	GO TO (1, 2, 3, 4, 5, 6, 7, 8, 9, 10, 11, 12, 13, 14, 15),ICCODE		INPT 990
C	GEAR AND LOAD CARD		INPT1000
1	READ (1,270) LGRTYP, LDIND, STLN2, PSI, THTA, PHI		INPT1010
	GO TO 150		INPT1020
C	FRICTION CARD		INPT1030
2	READ(1, 240) IDSPDR, SPDISP, COEFF		INPT1040
	GO TO 150		INPT1050
C	APPLIED DISPLACEMENT CARD		INPT1060
3	READ(1, 250) (XYZDSP(I), I=1,3)		INPT1070
	GO TO 150		INPT1080

C	STRUT MATERIAL CARDS	INPT1090
4	READ(1,260) IST, IMATL(IST)	INPT1100
	GO TO 150	INPT1110
C	SOLUTION PARAMETER CARD	INPT1120
5	READ(1,260) NSTEP, NITER, NMATL, IGLDIR, ENGABS, TOL	INPT1130
	GO TO 150	INPT1140
C	NODAL POINT CARDS	INPT1150
6	RFAD(1,240) NPN, (XYZPOS(I,NPN),I=1,3)	INPT1160
	GO TO 150	INPT1170
C	FOOTPAD CARD	INPT1180
7	READ(1,270) IFPATN, NFPCRL, (FPCRLD(I),I=1,3), (FPCRST(I),I=1,3)	INPT1190
	GO TO 150	INPT1200
C	MATERIAL PARAMETER CARDS	INPT1210
8	READ(1,270) MNO, INDPL(MNO), ELASBN(MNO), ELASAX(MNO),	INPT1220
	1 BINERT(MNO), AREA (MNO)	INPT1230
	GO TO 150	INPT1240
C	MATERIAL CRUSH CARDS	INPT1250
9	READ(1,230) MNO, IRET(MNO), IFIRST(MNO), SCMAX(MNO),	INPT1260
	1 STMAX(MNO), SRULC(MNO), SRULT(MNO)	INPT1270
	GO TO 150	INPT1280
C	COMPRESSION CRUSH DISTANCE CARDS	INPT1290
10	READ(1,240) MNO, (CDC(I,MNO),I=1,5)	INPT1300
	GO TO 150	INPT1310
C	TENSION CRUSH DISTANCE CARDS	INPT1320
11	READ(1,240) MNO, (CDT(I,MNO),I=1,5)	INPT1330
	GO TO 150	INPT1340
C	COMPRESSION SPRING RATE CARDS	INPT1350
12	READ(1,240) MNO, (SRC(I,MNO),I=1,5)	INPT1360
	GO TO 150	INPT1370
C	TENSION SPRING RATE CARDS	INPT1380
13	READ(1,240) MNO, (SRT(I,MNO),I=1,5)	INPT1390
	GO TO 150	INPT1400
C	COMPRESSION PLASTIC FORCE CARDS	INPT1410
14	READ(1,240) MNO, (PFC(I,MNO),I=1,5)	INPT1420
	GO TO 150	INPT1430
C	TENSION PLASTIC FORCE CARDS	INPT1440
15	READ(1,240) MNO, (PFT(I,MNO),I=1,5)	INPT1450
	GO TO 150	INPT1460
1000	CONTINUE	INPT1470
C	CONVERT EULER ANGLES TO RADIANS	INPT1480
	PHI = PHI / DEGRAD	INPT1490
	THTA = THTA / DEGRAD	INPT1500
	PSI = PSI / DEGRAD	INPT1510
C	SET DISPLACEMENT STEP NUMBER	INPT1520
	NDISPS = 0	INPT1530
	IF (LDIND .EQ. 2) GO TO 410	INPT1540
C		INPT1550
C	SET FRICTION PLANE INDICATOR IDSPD1 AND IDSPD2	INPT1560
C		INPT1570
	IDSPD1 = 0	INPT1580
	DO 400 I = 1, 3	INPT1590
	IF (I .EQ. IDSPDR) GO TO 400	INPT1600
	IDSPD2 = I	INPT1610
	IF (IDSPD1 .NE. 0) GO TO 400	INPT1620
	IDSPD1 = I	INPT1630
400	CONTINUE	INPT1640

C	SET NORMAL DISPLACEMENT	INPT1650
	XYZDSP(IDSPDR) = SPDISP	INPT1660
	410 CONTINUE	INPT1670
C		INPT1680
C	SET APPLIED DISPLACEMENT INCREMENTS	INPT1690
C		INPT1700
	DO 420 I = 1, 3	INPT1710
	420 XYZDS(I) = XYZDSP(I) / NSTEP	INPT1720
C	INITIALIZE THE ENERGY SUBROUTINE	INPT1730
	CALL ENERGI	INPT1740
	WRITE(6,320)	INPT1750
	320 FORMAT (1H1, 45X 36HINITIAL CONDITIONS FOR GEAR ANALYSIS ///	INPT1760
C		INPT1770
C	GET THE LANDER TO SURFACE COORDINATE TRANSFORMATION MATRIX,DC	INPT1780
C		INPT1790
	CALL GEOM	INPT1800
C		INPT1810
C	CONVERT NODE COORDINATES TO SURFACE COORDINATES	INPT1820
C		INPT1830
	NPOINT= 4	INPT1840
	IF (LGRTYP .EQ. 2) NPOINT= 5	INPT1850
	DO 700 I = 1, NPOINT	INPT1860
	DO 650 J = 1,3	INPT1870
	SAV(J) = 0.0	INPT1880
	DO 650 K = 1,3	INPT1890
	650 SAV(J) = SAV(J) + DC(J,K)*XYZPOS(K,I)	INPT1900
	DO 700 J = 1,3	INPT1910
	IF (ABS(SAV(J)) .GT. 1.E-10) GO TO 690	INPT1920
	XYZPOS(J,I) = .0	INPT1930
	GO TO 700	INPT1940
	690 XYZPOS(J,I) = SAV(J)	INPT1950
	700 CONTINUE	INPT1960
	RETURN	INPT1970
	230 FORMAT (5X 3I5, 4E10.5)	INPT1980
	240 FORMAT (5X I5, 10X 5E10.5)	INPT1990
	250 FORMAT (5X 15X 4E10.5)	INPT2000
	260 FORMAT (5X 4I5, 5X 2E10.5)	INPT2010
	270 FORMAT (5X 2I5, 5X 6E10.5)	INPT2020
	END	INPT2030

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SUBROUTINE GEOM
COMMON / CMAIN /
1  LGRTYP, LDIND , ITER, NPOINT, NREFP , NELEM , NUMB ,
2  NSTEP , NITER , TOL , ENGABS, IFPATN, NFPCRL, FPCRLD(3),
3  IGLDIR, NMATL , IST , IMATL(4) , MNO , FPCRST(3),
4  INDPL(4) , ELASBN(4) , ELASAX(4) ,
5  BINERT(4) , AREA(4) , IRET(4) ,
6  IFIRST(4) , SCMAX(4) , STMAX(4) ,
7  SRULC(4) , SRULT(4), CDC(5,4), SRC(5,4) ,
8  CDT(5,4) , CRC(5,4) , SRT(5,4) ,
9  PFC(5,4) , PFT(5,4) , NPN , XYZPOS(3,5)
9 , DC(3,3) , PHI , THTA , PSI
9 , CGYCDL(3), EGYCDL(3)
COMMON / CMAIN /
A  NITERS, NDISPS, STLN2 ,
B  IDSPDR, IDSPD1, IDSPD2, SPDISP, COEFF , XYZDSP(3),
C  STLN(4) , STLNO(4) , STRP(4) ,
D  SPNGC(4) , SPNGT(4) , DISTC(4) ,
E  DISTT(4) , PFORC (4) , PFORT(4) ,
F  FOREVC(4) , FOREVT(4) , IPOSC(4) ,
G  IPOST(4) , INDULC(4) , INDULT(4) ,
H  STR(4) , AXIALF(4) , AXSTIF(4) ,
I  IPREV(4) , XYZDS (3) , SFORCE(30) ,
J  CFORCE(30) , SUMFM4(6) , R24XYZ(3) ,
K  TM(9,4) , FMVCTL(12,4) , FMVCTG(12,4) ,
L  YSHRFP(4), ZSHRFP(4), YSHRMP(4), ZSHRMP(4), CGYXYZ(3),
M  YSHRFQ(4), ZSHRFQ(4), YSHRMQ(4), ZSHRMQ(4), EGYXYZ(3)
*****
CALCULATE THE DC TRANSFORMATION MATRIX FOR CONVERTING
LANDER COORDINATES TO SURFACE COORDINATES
*****
CALCULATE THE TRIG. FUNCTIONS ONLY ONCE IN GEOM
*****
COSPFI = COS( PHI )
SINPHI = SIN( PHI )
COSTHA = COS( THTA )
SINTHA = SIN( THTA )
COSPSI = COS( PSI )
SINPSI = SIN( PSI )
DC(1,1) = COSTHA * COSPSI
A = SINTHA * COSPSI
B = COSPHI * SINPSI
DC(2,1) = SINPHI * A - B
C = SINPHI * SINPSI
DC(3,1) = COSPHI * A + C
DC(1,2) = COSTHA * SINPSI
DC(2,2) = SINTHA * C + COSPHI * COSPSI
DC(3,2) = SINTHA * B - SINPHI * COSPSI
DC(1,3) = - SINTHA
DC(2,3) = SINPHI * COSTHA
DC(3,3) = COSPHI * COSTHA
*****
RETURN
END

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GEOM 10
GEOM 20
GEOM 30
GEOM 40
GEOM 50
GEOM 60
GEOM 70
GEOM 80
GEOM 90
GEOM 100
GEOM 110
GEOM 120
GEOM 130
GEOM 140
GEOM 150
GEOM 160
GEOM 170
GEOM 180
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GEOM 360
GEOM 370
GEOM 380
GEOM 390
GEOM 400
GEOM 410
GEOM 420
GEOM 430
GEOM 440
GEOM 450
GEOM 460
GEOM 470
GEOM 480
GEOM 490
GEOM 500
GEOM 510
GEOM 520
GEOM 530
GEOM 540

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C
C
C
C
C
C
C

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C

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PROGRAM INVTRP
DIMENSION TSTIFM(3,3)
DIMENSION STIFM(6,6), CDLXYZ (6)
COMMON / CMAIN /
1  LGRTYP, LDIND , ITER, NPOINT, NREFP , NELEM , NUMB ,
2  NSTEP , NITER , TOL , ENGABS, IFPATN, NFPCRL, FPCRLD(3),
3  IGLDIR, NMATL , IST , IMATL(4) , MNO , FPCRST(3),
4  INDPL(4) , ELASBN(4) , ELASAX(4) ,
5  BINERT(4) , AREA(4) , IRET(4) ,
6  IFIRST(4) , SCMAX(4) , STMAX(4) ,
7  SRULC(4) , SRULT(4), CDC(5,4), SRC(5,4) ,
8  CDT(5,4) , CRC(5,4) , SRT(5,4) ,
9  PFC(5,4) , PFT(5,4) , NPN , XYZPOS(3,5)
9  , DC(3,3) , PHI , THTA , PSI
9  , CGYCDL(3), EGYCDL(3)
COMMON / CMAIN /
A  NITERS, NDISPS, STLN2 ,
B  IDSPDR, IDSPD1, IDSPD2, SPDISP, COEFF , XYZDSP(3),
C  STLN(4) , STLNO(4) , STRP(4) ,
D  SPNGC(4) , SPNGT(4) , DISTC(4) ,
E  DISTT(4) , PFORC(4) , PFORT(4) ,
F  FOREVC(4) , FOREVT(4) , IPOS(4) ,
G  IPOST(4) , INDULC(4) , INDULT(4) ,
H  STR(4) , AXIALF(4) , AXSTIF(4) ,
I  IPREV(4) , XYZDS (3) , SFORCE(30) ,
J  CFCRCE(30) , SUMFM4(6) , R24XYZ(3) ,
K  TM(36) , FMVCTL(12,4) , FMVCTG(12,4) ,
L  YSHRFP(4), ZSHRFP(4), YSHRMP(4), ZSHRMP(4), CGYXYZ(3),
M  YSHRFQ(4), ZSHRFQ(4), YSHRMQ(4), ZSHRMQ(4), EGYXYZ(3)
NITERS = 0
NELEM=3
NITERC = 1
ICOS = 10 + (IDSPDR-1) * 3
IFIN = 0
NITERS = 0
C
C  CALCULATE INITIAL STRUT LENGTHS AND LOCAL COORDINATE TRANS M.
CALL TRANSM(XYZPOS(1,1),XYZPOS(1,4),XYZPOS(1,2),TM( 1),STLNO(1) )
CALL TRANSM(XYZPOS(1,2),XYZPOS(1,4),XYZPOS(1,3),TM(10),STLNO(2) )
CALL TRANSM(XYZPOS(1,3),XYZPOS(1,4),XYZPOS(1,2),TM(19),STLNO(3) )
C  SAVE MAIN STRUT LENGTH
STLNS = STLNO(2)
DO 110 I = 1, 3
110 CDLXYZ(I) = XYZDS(I)
DO 120 I=1,3
120 STLN(I)=STLNO(I)
CALL OUTPT1
130 CONTINUE
C
C  UPDATE FOOTPAD JOINT COORDINATES
C
DO 140 I = 1, 3

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140 XYZPOS(I,NPOINT) = XYZPOS(I,NPOINT) + CDLXYZ(I)          INVT 530
    IF ( NITERS .LE. 1 ) GO TO 145                          INVT 540
    RMOVE = SQRT((BASEPX-XYZPOS(1,4))**2 + (BASEPY-XYZPOS(2,4))**2 INVT 550
    1      + (BASEPZ-XYZPOS(3,4))**2 )                      INVT 560
    IF ( RMOVE .LT. RADRAG ) GO TO 145                      INVT 570
    WRITE(6,144)                                           INVT 580
144 FORMAT (43HIERROR *** NO SOLUTION FOUND AT THIS LEVEL. ) INVT 590
    CALL OUTPT1                                           INVT 600
    RETURN                                               INVT 610
145 CONTINUE                                             INVT 620
C      CALCULATE CURRENT STRUT LENGTH AND GET TRANS. M.    INVT 630
    CALL TRANSM(XYZPOS(1,1),XYZPOS(1,4),XYZPOS(1,2),TM( 1),STLN( 1) ) INVT 640
    CALL TRANSM(XYZPOS(1,2),XYZPOS(1,4),XYZPOS(1,3),TM(10),STLN( 2) ) INVT 650
    CALL TRANSM(XYZPOS(1,3),XYZPOS(1,4),XYZPOS(1,2),TM(19),STLN( 3) ) INVT 660
C
C      CALCULATE STRUT FORCES AND AXIAL STIFFNESSES      INVT 670
C
C
C      NITERS = NITERS + 1                                INVT 700
    ISTOP1 = 0                                           INVT 705
    DO 170 IST=1, NELCM                                  INVT 710
C      COMPUTE STROKE                                    INVT 720
    STR(IST) = STLN(IST) - STLNO(IST)                   INVT 730
C      GET THE MATERIAL NO.                              INVT 740
    MNO = IMATL(IST)                                    INVT 750
    IF ( INDPL(MNO) .EQ. 2 ) GO TO 160                   INVT 760
C      COMPUTE AXIAL FORCE AND STIFFNESS - ELASTIC MATERIAL IN STRUT INVT 770
    AXSTIF(IST) = AREA(MNO)*ELASAX(MNO)/STLN(IST)      INVT 780
    AXIALF(IST) = STR(IST)*AXSTIF(IST)                  INVT 790
    GO TO 165                                           INVT 800
C      COMPUTE AXIAL FORCE AND STIFFNESS - PLASTIC MATERIAL IN STRUT INVT 810
160 CONTINUE                                           INVT 820
    CALL STRUT ( IST, SPNGC(IST) , SPNGT(IST) , DISTC(IST) , INVT 830
    1      DISTT(IST) , PFORC(IST) , PFORT(IST) ,      INVT 840
    2      FOREVC(IST) , FOREVT(IST) , INDULC(IST) ,  INVT 850
    3      INDULT(IST) , IPREV(IST) , STRP(IST) ,      INVT 860
    4      IPOSC(IST) , IPOST(IST) , IRET(MNO) ,      INVT 870
    5      SCMAX(MNO) , STMAX(MNO) , CDC(1,MNO) ,      INVT 880
    6      SRULT(MNO) , SRULC(MNO) , SRT(1,MNO) ,      INVT 890
    7      SRC(1,MNO) , PFT(1,MNO) , PFC(1,MNO) ,      INVT 900
    8      STR(IST) , AXIALF(IST) , AXSTIF(IST) ,      INVT 910
    9      IFIRST(MNO) , NITERS , CDT(1,MNO) ,        INVT 920
    A      ISTOP2 )                                     INVT 925
    IF ( ISTOP2 .LT. 0 ) ISTOP1 = 1                     INVT 930
165 CONTINUE                                           INVT 940
C      SET LOCAL FORCE VECTOR WITH AXIAL FORCES          INVT 950
    FMVCTL(1,IST) = -AXIALF(IST)                        INVT 960
    FMVCTL(4,IST) = AXIALF(IST)                        INVT 970
C      CONVERT LOCAL FORCE VECTOR TO GLOBAL FORCE VECTOR  INVT 980
    CALL TRALMG( FMVCTG(1,IST) , FMVCTL(1,IST) , TM( 9*IST-8) , 2 ) INVT 990
170 CONTINUE                                           INVT1000
    IF ( ISTOP1 .EQ. 1 ) RETURN                         INVT1005
C
C      SUM FORCES AT NODE POINT FOUR                    INVT1010
C
C
C      DO 180 I = 1, 3                                   INVT1040
    SUMFM4(I) = 0.0                                     INVT1050

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      K = I + 3
      DO 180 J = 1, NELEM
180 SUMFM4(I) = SUMFM4(I) + FMVCTG(K,J)
C
C      IF THREE FOOTPAD JOINT DISPLACEMENTS ARE APPLIED, GO TO
C      OUTPUT AND ENERGY SECTION
C
C      IF ( LDIND .EQ. 2 ) GO TO 400
C      SKIP CONVERGENCE CHECK ON FIRST PASS
C      CHECK FOR CONVERGENCE
C      END OF LOOP
C      IF ( IFIN .EQ. 1 ) GO TO 400
C      IF ( NITERS.GT. NITER ) GO TO 400
C      COMPUTE FRICTION FORCE
      FRICTF = COEFF * SUMFM4( IDSPDR )
      ABSFIR = SQRT( SUMFM4(IDSPD1)**2 + SUMFM4(IDSPD2)**2 )
      IF ( NITERS .NE. NITERC ) GO TO 190
      IF ( ABSFIR .LE. AES( FRICTF ) ) GO TO 400
C      SET NORMAL FOOTPAD JOINT DISPLACEMENT INCREMENT TO ZERO
      CDLXYZ(IDSPDR) = 0.0
C      PICK UP THE COS OF THE ANGLE (ALF) BETWEEN THE MAIN STRUT AND
C      THE NORMAL
      COSALF = ABS(TM( ICOS) )
      IF ( COSALF .LT. 1. ) GO TO 500
      WRITE(6,501)
501 FORMAT ( 100H1 ERROR *** DATA CASE TERMINATED, ANGLE BETWEEN MAIN
1 STRUT AND THE NORMAL TO THE LANDING SURFACE IS )
      WRITE(6,502)
502 FORMAT ( 60X 14H.EQ. 0 DEGREES )
      CALL OUTPT1
      RETURN
500 CONTINUE
      IF ( COSALF .GT. 0. ) GO TO 505
      WRITE(6,501)
      WRITE(6,503 )
503 FORMAT ( 60X 15H.EQ. 90 DEGREES )
      CALL OUTPT1
      RETURN
505 CONTINUE
      TANALF = SQRT( 1. - COSALF**2 ) / COSALF
      RADLIM = AMAX1( XYZDS(IDSPDR) / TANALF, XYZDS(IDSPDR) * TANALF )
      RADLIM = AMIN1(RADLIM,SQRT(STLNS**2-(STLNS-XYZDS(IDSPDR))**2))
C      SET OUTER RADIUS LIMIT
      RADRAG = RADLIM * 1.05
      RDSTEP = RADRAG / 100.
C      DETERMINE THE CENTER OF SOLUTION CIRCLE
      BASEPX = XYZPOS(1,4)
      BASEPY = XYZPOS(2,4)
      BASEPZ = XYZPOS(3,4)
190 CONTINUE
C      RDSTEP IS POSITIVE AND DELTF2 NEGATIVE UNTIL SOLUTION
C      POINT IS PASSED
      DELTF2 = ABS( FRICTF ) - ABSFIR
C      TEST FOR CONVERGENCE
      IF ( ABS(DELTF2) .LE. .0005*ABS(FRICTF) ) GO TO 400
      IF ( RDSTEP .LT. 0.0 ) GO TO 235

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	IF (DELTF2 .LT. 0.0) GO TO 230	INVT1600
C	CALCULATE BACK STEP TO SOLUTION	INVT1610
	RDSTPS = RDSTEP	INVT1615
	RDSTEP = - DELTF2*RDSTEP / (ABS(DELTF1) + DELTF2)	INVT1620
C	SAVE DIFFERENCE IN RDSTEP	INVT1622
	RDSTPS = - (RDSTEP + RDSTPS)*.5	INVT1624
	GO TO 230	INVT1630
235	CONTINUE	INVT1640
C	ONCE SOLUTION POINT IS PASSED, CONTINUE TAKING BACK STEPS	INVT1650
C	UNTIL DELTF2 AGAIN BECOMES NEGATIVE AFTER WHICH ONE	INVT1660
C	MORE FORWARD STEP IS TAKEN	INVT1670
	RDSTEP = RDSTPS	INVT1680
	IF (DELTF2 .GT. 0.0) GO TO 230	INVT1690
	RDSTEP = DELTF2 * RDSTEP / (ABS(DELTF2) + DELTF1)	INVT1700
	IFIN=1	INVT1715
230	CONTINUE	INVT1720
	COS1 = SUMFM4(IDSPD1) / ABSFIR	INVT1730
	COS2 = SUMFM4(IDSPD2) / ABSFIR	INVT1740
240	CONTINUE	INVT1750
C	COMPUTE NEXT SLIDING DISPLACEMENT INCREMENTS	INVT1760
	CDLXYZ(IDSPD1) = - COS1 * RDSTEP	INVT1770
	CDLXYZ(IDSPD2) = - COS2 * RDSTEP	INVT1780
	XYZDS(IDSPD1) = XYZDS(IDSPD1) + CDLXYZ(IDSPD1)	INVT1790
	XYZDS(IDSPD2) = XYZDS(IDSPD2) + CDLXYZ(IDSPD2)	INVT1800
C	SAVE FRICTION FORCE UNBALANCE	INVT1810
	DELTF1 = DELTF2	INVT1820
	GO TO 130	INVT1830
400	CONTINUE	INVT1840
C		INVT1850
C	OUTPUT AND ENERGY SECTION	INVT1860
C		INVT1870
	CALL ENERGY(SUMFM4)	INVT1880
	CALL OUTPUT	INVT1890
	IF (NITERS .LE. NITER) GO TO 405	INVT1900
	WRITE(6,403)	INVT1910
403	FORMAT (34H1MAXIMUM ITERATION NUMBER EXCEEDED)	INVT1920
	RETURN	INVT1930
405	CONTINUE	INVT1940
	NITERS = 0	INVT1950
	IF (NDISPS .GE. NSTEP) RETURN	INVT1960
	IF (IFPATN .EQ. 1) GO TO 425	INVT1970
	IF (LDIND .EQ. 2) GO TO 130	INVT1980
425	CONTINUE	INVT1990
	IF (SUMFM4(IDSPDR) .GE. 0.0) GO TO 428	INVT2000
	WRITE(6,426)	INVT2010
426	FORMAT (21H1NEGATIVE NORMAL LOAD)	INVT2020
	RETURN	INVT2030
428	CONTINUE	INVT2040
	IF (LDIND .EQ. 2) GO TO 130	INVT2050
	NITERS = 1	INVT2060
	XYZDS(IDSPD1) = 0.0	INVT2070
	XYZDS(IDSPD2) = 0.0	INVT2080
	DO 430 I = 1, 3	INVT2090
430	CDLXYZ(I) = XYZDS(I)	INVT2100
C	SAVE MAIN STRUT LENGTH	INVT2110
	STLNS = STLN(2)	INVT2120

	NITERC = 2		INVT2130
	IFIN = 0		INVT2140
C			INVT2150
C	SET STRUT SAVE VARIABLES FOR THIS POSITION		INVT2160
C			INVT2170
	ISTOP1 = 0		INVT2175
	DO 440 IST = 1, 3		INVT2180
	MNO = IMATL(IST)		INVT2190
	IF (INDPL(MNO) .NE. 2) GO TO 440		INVT2200
	CALL STRUT (IST, SPNGC(IST), SPNGT(IST), DISTC(IST),		INVT2210
1	DISTT(IST), PFORC(IST), PFORT(IST),		INVT2220
2	FOREVC(IST), FOREVT(IST), INDULC(IST),		INVT2230
3	INDULT(IST), IPREV(IST), STRP(IST),		INVT2240
4	IPOSC(IST), IPOST(IST), IRET(MNO),		INVT2250
5	SCMAX(MNO), STMAX(MNO), CDC(1,MNO),		INVT2260
6	SRULT(MNO), SRULC(MNO), SRT(1,MNO),		INVT2270
7	SRC(1,MNO), PFT(1,MNO), PFC(1,MNO),		INVT2280
8	STR(IST), AXIALF(IST), AXSTIF(IST),		INVT2290
9	IFIRST(MNO), NITERS, CDT(1,MNO),		INVT2300
A	ISTOP2)		INVT2305
	IF (ISTOP2 .LT. 0) ISTOP1 = 1		INVT2310
440	CONTINUE		INVT2320
	IF (ISTOP1 .EQ. 1) RETURN		INVT2325
	GO TO 130		INVT2330
	END		INVT2340

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PROGRAM CANTIL
COMMON / CMAIN /
1  LGRTYP, LDIND , INITER, NPOINT, NREFP , NELEM , NUMB ,
2  NSTEP , NITER , TOL , ENGABS, IFPATN, NFPCRL, FPCRLD(3),
3  IGLDIR, NMATL , IST , IMATL(4) , MNO , FPCRST(3),
4  INDPL(4) , ELASBN(4) , ELASAX(4) ,
5  BINERT(4) , AREA(4) , IRET(4) ,
6  IFIRST(4) , SCMAX(4) , STMAX(4) ,
7  SRULC(4) , SRULT(4), CDC(5,4), SRC(5,4) ,
8  CDT(5,4) , CRC(5,4) , SRT(5,4) ,
9  PFC(5,4) , PFT(5,4) , NPN , XYZPOS(3,5)
9 , DC(3,3) , PHI , THTA , PSI
9 , CGYCDL(3), EGYCDL(3)
COMMON / CMAIN /
A  NITERS, NDISPS, STLN2 ,
B  IDSPDR, IDSPD1, IDSPD2, SPDISP, COEFF , XYZDSP(3),
C  STLN(4) , STLNO(4) , STRP(4) ,
D  SPNGC(4) , SPNGT(4) , DISTC(4) ,
E  DISTT(4) , PFORC(4) , PFORT(4) ,
F  FOREVC(4) , FOREVT(4) , IPOSC(4) ,
G  IPOST(4) , INDULC(4) , INDULT(4) ,
H  STR(4) , AXIALF(4) , AXSTIF(4) ,
I  IPREV(4) , XYZDS(3) , SFORCE(30) ,
J  CFORCE(30) , SUMFM4(6) , R24XYZ(3) ,
K  TM(9,4) , FMVCTL(12,4) , FMVCTG(12,4) ,
L  YSHRFP(4), ZSHRFP(4), YSHRMP(4), ZSHRMP(4), CGYXYZ(3),
M  YSHRFQ(4), ZSHRFQ(4), YSHRMO(4), ZSHRMO(4), EGYXYZ(3)
DIMENSION TSTIFM(15,15)
DIMENSION CDLXYZ(15)
DIMENSION STIFM6(6,6), STIFM(12,12), SARRY(15)
DOUBLE PRECISION CDL(12),SAVED(78),TRAC(12)
NITERS = 0
I1ST = 0
I2ND = 0
IFIN = 0
NELEM = 4
ICOS = 1 + (IDSPDR-1) * 3
IFRIST = 0
C  CALCULATE NODE POINT FOUR COORDINATES
TL25 = 0.0
DO 100 I = 1, 3
XYZPOS(I,4) = XYZPOS(I,5) - XYZPOS(I,2)
TL25 = TL25 + XYZPOS(I,4)**2
100 CONTINUE
TL25 = SORT( TL25 )
DO 105 I = 1, 3
R24XYZ(I) = XYZPOS(I,1)
105 XYZPOS(I,4) = XYZPOS(I,2) + STLN2* XYZPOS(I,4)/ TL25
C  CALCULATE INITIAL STRUT LENGTHS AND LOCAL COORDINATE
C  TRANSFORMATION MATRICES
CALL TRANSX(XYZPOS(1,1),XYZPOS(1,2),TM(1,1),STLNO(1) )
CALL TRANSX(XYZPOS(1,2),XYZPOS(1,4),R24XYZ( 1 ),TM(1,2),STLNO(2) )

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CALL TRANSM(XYZPOS(1,3),XYZPOS(1,4),XYZPOS(1,2),TM(1,3),STLNO(2) )CANT 530
CALL TRANSM(XYZPOS(1,5),XYZPOS(1,4),R24XYZ( 1 ),TM(1,4),STLNO(4) )CANT 540
DO 120 I = 1, 3 CANT 550
120 CDLXYZ(I) = XYZDS(I) CANT 560
DO 125 I = 4, 15 CANT 570
125 CDLXYZ(I) = 0.0 CANT 580
DO 130 I = 1,4 CANT 590
130 STLN(I) = STLNO(I) CANT 600
C SAVE TOTAL MAIN STRUT LENGTH = STRUT 2 + STRUT 4 CANT 610
STLNS = STLN(2) + STLN(4) CANT 620
C CANT 630
C PRINT INITIAL POSITIONS CANT 640
C CANT 650
CALL OUTPT1 CANT 660
GO TO 150 CANT 670
1000 CONTINUE CANT 680
C CANT 690
C UPDATE COORDINATES OF NODE POINTS FOUR AND FIVE CANT 700
C CANT 710
DO 140 I = 1, 3 CANT 720
K = I + 3 CANT 730
XYZPOS(I,NPOINT) = XYZPOS(I,NPOINT) + CDLXYZ(I) CANT 740
XYZPOS(I, 4 ) = XYZPOS(I, 4 ) + CDLXYZ(K) CANT 750
140 CONTINUE CANT 760
IF ( NITERS .LE. 1 ) GO TO 145 CANT 770
IF ( I2ND .NE. 1 ) GO TO 147 CANT 780
I2ND = 0 CANT 790
CDLXYZ (1) = 0.0 CANT 800
CDLXYZ (2) = 0.0 CANT 810
CDLXYZ (3) = 0.0 CANT 820
RMOVE = SQRT((BASEPX-XYZPOS(1,5))**2 + (BASEPY-XYZPOS(2,5))**2 CANT 830
1 + (BASEPZ-XYZPOS(3,5))**2 ) CANT 840
IF ( RMOVE .LT. RADRAG ) GO TO 147 CANT 850
WRITE(6,144) CANT 860
144 FORMAT (43H1ERROR *** NO SOLUTION FOUND AT THIS LEVEL. ) CANT 870
CALL OUTPT1 CANT 880
RETURN CANT 890
145 CONTINUE CANT 900
IF ( LDIND .EQ. 2 ) GO TO 146 CANT 910
C DETERMINE THE CENTER OF SOLUTION CIRCLE CANT 920
BASEPX = XYZPOS(1,5) CANT 930
BASEPY = XYZPOS(2,5) CANT 940
BASEPZ = XYZPOS(3,5) CANT 950
146 CDLXYZ (1) = 0.0 CANT 960
CDLXYZ (2) = 0.0 CANT 970
CDLXYZ (3) = 0.0 CANT 980
147 CONTINUE CANT 990
C CANT1000
C CALCULATE BENDING MOMENTS AND SHEAR FORCES CANT1010
C CANT1020
CALL BNDLDS CANT1030
C SET LOCAL FORCE/MOMENT VECTOR WITH BENDING LOADS CANT1040
FMVCTL( 2,2)= YSHRFP(2) CANT1050
FMVCTL( 5,2)= YSHRFQ(2) CANT1060
FMVCTL(12,2)= ZSHRMQ(2) CANT1070
FMVCTL( 2,4) = YSHRFP(4) CANT1080

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      FMVCTL( 5,4) = YSHRFQ(4)                                CANT1090
      FMVCTL(12,4) = ZSHRMQ(4)                                CANT1100
C      CALCULATE CURRENT STRUT LENGTHS AND TRANSFORMATION MATRICES CANT1110
      CALL TRANSM(XYZPOS(1,1),XYZPOS(1,4),XYZPOS(1,2),TM(1,1),STLN(1) ) CANT1120
      CALL TRANSM(XYZPOS(1,2),XYZPOS(1,4),R24XYZ( 1 ),TM(1,2),STLN(2) ) CANT1130
      CALL TRANSM(XYZPOS(1,3),XYZPOS(1,4),XYZPOS(1,2),TM(1,3),STLN(3) ) CANT1140
      CALL TRANSM(XYZPOS(1,5),XYZPOS(1,4),R24XYZ( 1 ),TM(1,4),STLN(4) ) CANT1150
150 CONTINUE                                                  CANT1160
C
C      CALCULATE STRUT FORCES AND AXIAL STIFFNESSES           CANT1170
C
C      NITERS = NITERS + 1                                     CANT1180
C      ISTOP1 = 0                                             CANT1190
C      DO 190 IST = 1, NELEM                                    CANT1200
C          COMPUTE STROKE                                       CANT1205
C          STR(IST) = STLN(IST) - STLNO(IST)                    CANT1210
C          GET MATERIAL NO.                                     CANT1215
C          MNO = IMATL(IST)                                     CANT1220
C          IF ( INDPL(MNO) .EQ. 2 ) GO TO 160                    CANT1225
C          COMPUTE AXIAL FORCE AND STIFFNESS----ELASTIC MATERIAL IN STRUTCANT1230
C          AXSTIF(IST) = AREA(MNO)*ELASAX(MNO) / STLN(IST)     CANT1235
C          AXIALF(IST) = STR(IST) * AXSTIF(IST)                 CANT1240
C          GO TO 170                                             CANT1245
C          COMPUTE AXIAL FORCE AND STIFFNESS----PLASTIC MATERIAL IN STRUTCANT1250
160 CONTINUE                                                  CANT1260
      CALL STRUT ( IST, SPNGC(IST) , SPNGT(IST) , DISTC(IST) ,
1          DISTT(IST) , PFORC(IST) , PFORT(IST) ,              CANT1265
2          FOREVC(IST) , FOREVT(IST) , INCULC(IST) ,           CANT1270
3          INDULT(IST) , IPREV(IST) , STRP(IST) ,              CANT1275
4          IPOSC(IST) , IPOST(IST) , IRET(MNO) ,              CANT1280
5          SCMAX(MNO) , STMAX(MNO) , CDC(1,MNO) ,              CANT1285
6          SRULT(MNO) , SRULC(MNO) , SRT(1,MNO) ,              CANT1290
7          SRC(1,MNO) , PFT(1,MNO) , PFC(1,MNO) ,              CANT1295
8          STR(IST) , AXIALF(IST) , AXSTIF(IST) ,              CANT1300
9          IFIRST(MNO) , NITERS , CDT(1,MNO) ,                  CANT1305
A          ISTOP2 )                                             CANT1310
      IF ( ISTOP2 .LT. 0 ) ISTOP1 = 1                           CANT1315
170 CONTINUE                                                  CANT1320
C      SET LOCAL FORCE/MOMENT VECTOR WITH AXIAL FORCES         CANT1325
C      FMVCTL(1,IST) = -AXIALF(IST)                             CANT1330
C      FMVCTL(4,IST) = AXIALF(IST)                              CANT1335
190 CONTINUE                                                  CANT1340
      IF ( ISTOP1 .EQ. 1 ) RETURN                                CANT1345
C
C      CONVERT LOCAL FORCE/MOMENT VECTORS TO GLOBAL FORCE/MOMENT CANT1350
C      VECTORS                                                 CANT1355
C
C      DO 210 IST = 1, NELEM                                    CANT1360
C          CALL TRALMG( FMVCTG(1,IST), FMVCTL(1,IST), TM(1,IST), 4 ) CANT1365
210 CONTINUE                                                  CANT1370
C
C      SUM FORCE/MOMENTS AT NODE POINT FOUR                     CANT1375
C      AND-SET UP SOLUTION ARRAY                                CANT1380
C
C      DO 230 I = 1, 3                                          CANT1385
C          L = I + 9                                             CANT1390
C          K = I + 3                                             CANT1395
CANT1400

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	SUMOK1=FMVCTG(K,1)+FMVCTG(K,3)	CANT1610
	SUMOK2=FMVCTG(K,2)+FMVCTG(K,4)	CANT1620
	SUMFM4(I)=SUMOK1+SUMOK2	CANT1630
	SUMOK1=FMVCTG(L,1)+FMVCTG(L,3)	CANT1640
	SUMOK2=FMVCTG(L,2)+FMVCTG(L,4)	CANT1650
	SUMFM4(K)=SUMOK1+SUMOK2	CANT1660
	SARRY (K)= SUMFM4(I)	CANT1670
	SARRY (L)= SUMFM4(K)	CANT1680
	SARRY (I)= FMVCTG(I,4)	CANT1690
230	CONTINUE	CANT1700
	IF (IFRIST .EQ. 0) GO TO 280	CANT1710
C	GET MOMENTS AT NODES TWO AND FIVE FOR SOLUTION ARRAY	CANT1720
	DO 240 I = 7,9	CANT1730
	SARRY (I+6) = FMVCTG(I,2)	CANT1740
	SARRY (I) = FMVCTG(I,4)	CANT1750
240	CONTINUE	CANT1760
C	END OF LOOP	CANT1770
	IF (NITERS .GT. NITER) GO TO 400	CANT1780
	DO 260 I = 4, 15	CANT1790
	CDLXYZ(I) = -SARRY(I)	CANT1800
260	CONTINUE	CANT1810
C	CHECK FOR CONVERGENCE	CANT1820
	DO 265 I = 4, 15	CANT1830
	IF (ABS(CDLXYZ(I)) .GT. TOL) GO TO 280	CANT1840
265	CONTINUE	CANT1850
C	IF THREE FOOTPAD JOINT DISPLACEMENTS ARE APPLIED,	CANT1860
C	GO TO OUTPUT AND ENERGY SECTION	CANT1870
	IF (LDIND .EQ. 2) GO TO 400	CANT1880
	IF (IFIN .EQ. 1) GO TO 400	CANT1890
C	COMPUTE FRICTION FORCE	CANT1900
	FRICTF = COEFF * FMVCTG(IDSPDR,4)	CANT1910
	ABSFIR = SQRT(FMVCTG(IDSPD1,4)**2 + FMVCTG(IDSPD2,4)**2)	CANT1920
	IF (I1ST .NE. 0) GO TO 233	CANT1930
	I1ST = 1	CANT1940
	IF (ABSFIR .LE. ABS(FRICTF)) GO TO 400	CANT1950
C	PICK UP THE COS OF THE ANGLE (ALF) BETWEEN THE MAIN STRUT AND	CANT1960
C	THE NORMAL	CANT1970
	COSALF = ABS(TM(ICOS, 2))	CANT1980
	IF (COSALF .LT. 1.) GO TO 500	CANT1990
	WRITE(6,501)	CANT2000
501	FORMAT (100H1 ERROR *** DATA CASE TERMINATED, ANGLE BETWEEN MAIN	CANT2010
	1 STRUT AND THE NORMAL TO THE LANDING SURFACE IS)	CANT2020
	WRITE(6,502)	CANT2030
502	FORMAT (60X 14H.EQ. 0 DEGREES)	CANT2040
	CALL OUTPT1	CANT2050
	RETURN	CANT2060
500	CONTINUE	CANT2070
	IF (COSALF .GT. 0.) GO TO 505	CANT2080
	WRITE(6,501)	CANT2090
	WRITE(6,503)	CANT2100
503	FORMAT (60X 15H.EQ. 90 DEGREES)	CANT2110
	CALL OUTPT1	CANT2120
	RETURN	CANT2130
505	CONTINUE	CANT2140
	TANALF = SQRT(1. - COSALF**2) / COSALF	CANT2150
	RADLIM = AMAX1(XYZDS(IDSPDR) / TANALF, XYZDS(IDSPDR) * TANALF)	CANT2160

	RADLIM=AMINI(RADLIM,SQRT((STLNS**2-(STLNS-XYZDS(IDSPDR))**2))	CANT2170
C	SET OUTER RADIUS LIMIT	CANT2180
	RADRAG = RADLIM * 1.05	CANT2190
	RDSTEP = RADRAG / 50.	CANT2200
233	CONTINUE	CANT2210
C	RDSTEP IS POSITIVE AND DELTF2 NEGATIVE UNTIL SOLUTION POINT	CANT2220
C	IS PASSED	CANT2230
	DELTF2 = ABS(FRICTF) - ABSFIR	CANT2240
C	TEST FOR CONVERGENCE	CANT2250
	IF (ABS(DELTF2) .LE. .0005*ABS(FRICTF)) GO TO 400	CANT2260
	IF (RDSTEP .LT. 0.0) GO TO 235	CANT2270
	IF (DELTF2 .LT. 0.0) GO TO 237	CANT2280
C	CALCULATE BACK STEP TO SOLUTION	CANT2290
	RDSTPS = RDSTEP	CANT2300
	RDSTEP = - DELTF2*RDSTEP / (ABS(DELTF1) + DELTF2)	CANT2310
C	SAVE DIFFERENCE IN RDSTEP	CANT2320
	RDSTPS = -(RDSTEP + RDSTPS) * .5	CANT2330
	GO TO 237	CANT2340
235	CONTINUE	CANT2350
C	ONCE SOLUTION POINT IS PASSED, CONTINUE TAKING BACK STEPS	CANT2360
C	UNTIL DELTF2 AGAIN BECOMES NEGATIVE AFTER WHICH ONE	CANT2370
C	MORE FORWARD STEP IS TAKEN	CANT2380
	RDSTEP = RDSTPS	CANT2390
	IF (DELTF2 .GT. 0.0) GO TO 237	CANT2400
	RDSTEP = DELTF2*RDSTEP / (ABS(DELTF2) + DELTF1)	CANT2410
	IFIN=1	CANT2420
237	CONTINUE	CANT2430
	COS1 = FMVCTG(IDSPD1,4) / ABSFIR	CANT2440
	COS2 = FMVCTG(IDSPD2,4) / ABSFIR	CANT2450
C	COMPUTE NEXT SLIDING DISPLACEMENT INCREMENTS	CANT2460
	CDLXYZ(IDSPD1) = - COS1 * RDSTEP	CANT2470
	CDLXYZ(IDSPD2) = - COS2 * RDSTEP	CANT2480
C	SAVE FRICTION FORCE UNBALANCE	CANT2490
	DELTF1 = DELTF2	CANT2500
	I2ND = 1	CANT2510
	DO 239 I = 4, 15	CANT2520
239	CDLXYZ(I) = 0.0	CANT2530
280	CONTINUE	CANT2540
	IFRIST = 1	CANT2550
C	INITIALIZE THE SOLUTION STIFFNESS MATRIX	CANT2560
	DO 290 I = 13, 15	CANT2570
	DO 290 J = 1, 9	CANT2580
	TSTIFM(J,I) = 0.0	CANT2590
290	TSTIFM(I,J) = 0.0	CANT2600
C	COMPUTE THE SM ELEMENT FOUR	CANT2610
	IST = 4	CANT2620
	CALL STFMCF(STIFM)	CANT2630
	CALL TRNFSM(STIFM, TM(1,4), 12)	CANT2640
	DO 300 I = 4,12	CANT2650
	DO 300 J = 1, 12	CANT2660
	TSTIFM(I,J) = STIFM(I,J)	CANT2670
300	CONTINUE	CANT2680
C	COMPUTE THE SM. FOR ELEMENT TWO	CANT2690
	IST = 2	CANT2700
	CALL STFMCF(STIFM)	CANT2710
	CALL TRNFSM(STIFM, TM(1,2), 12)	CANT2720

```

DO 310 I = 4, 6
DO 310 J = 4, 6
K = J + 6
L = J + 3
M = J + 9
TSTIFM(I,J) = STIFM(I,J) + TSTIFM(I,J)
TSTIFM(I,K) = STIFM(I,K) + TSTIFM(I,K)
TSTIFM(I,M) = STIFM(I,L)
N = I + 6
TSTIFM(N,J) = STIFM(N,J) + TSTIFM(N,J)
TSTIFM(N,K) = STIFM(N,K) + TSTIFM(N,K)
TSTIFM(N,M) = STIFM(N,L)
N = I + 9
TSTIFM(N,J) = STIFM(I+3,J)
TSTIFM(N,K) = STIFM(I+3,K)
TSTIFM(N,M) = STIFM(I+3,L)
310 CONTINUE
C          COMPUTE THE SM. FOR ELEMENT ONE AND THREE
DO 320 IST = 1, 3, 2
CALL STFMIT ( STIFM6, AXIALF(IST), STLN(IST), AXSTIF(IST) )
CALL TRNFSM ( STIFM6, TM(1,IST), 6 )
DO 330 I = 4, 6
DO 330 J = 4, 6
TSTIFM(I,J) = TSTIFM(I,J)+STIFM6(I,J)
330 CONTINUE
320 CONTINUE
C          ALTER THE SOLUTION VECTOR FOR KNOWN DISPLACEMENTS AT NODE FIVE
DO 340 I = 4, 15
DO 340 J = 1, 3
340 CDLXYZ(I) = CDLXYZ(I) - CDLXYZ(J)*TSTIFM(I,J)
C          THE 12 BY 12 MATRIX STORED IN TSTIFM IN COLUMNS 4-15 AND
C          ROWS 4-15 IS THE STIFFNESS MATRIX OF INTEREST. STORE THE
C          UPPER TRIANGULAR PORTION OF THIS MATRIX IN SAVED
JJ=0
DO 50 I=4,15
DO 50 J=4,I
JJ=JJ+1
50 SAVED(JJ)=TSTIFM(J,I)
DO 1 I=1,12
1 CDL(I)=CDLXYZ(I+3)
CALL DMFSS(SAVED,12,1.E-12,IRANK,TRAC)
IF(IRANK.LE.0) GO TO 53
INC=100
GO TO 55
53 WRITE(6,54)IRANK
54 FORMAT(*1 BAD STIFFNESS MATRIX ----RANK = *,I5)
CALL OUTPT1
RETURN
55 CONTINUE
CALL DMLSS(SAVED,12,IRANK,TRAC,INC,CDL,IER)
DO 6 I=1,12
6 CDLXYZ(I+3)=CDL(I)
C
C          CORRECT FOR ROUND OFF ERROR
C
DO 360 I = 4,6

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CANT2730
CANT2740
CANT2750
CANT2760
CANT2770
CANT2780
CANT2790
CANT2800
CANT2810
CANT2820
CANT2830
CANT2840
CANT2850
CANT2860
CANT2870
CANT2880
CANT2890
CANT2900
CANT2910
CANT2920
CANT2930
CANT2940
CANT2950
CANT2960
CANT2970
CANT2980
CANT2990
CANT3000
CANT3010
CANT3020
CANT3030
CANT3040
CANT3050
CANT3060
CANT3070
CANT3080
CANT3090
CANT3100
CANT3110
CANT3120
CANT3130
CANT3140
CANT3150
CANT3160
CANT3170
CANT3180
CANT3190
CANT3200
CANT3210
CANT3220
CANT3230
CANT3240
CANT3250
CANT3260
CANT3270
CANT3280

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	IF (ABS(CDLXYZ(I)) .LT. 1.E-11) CDLXYZ(I) = 0.0	CANT3290
360	CONTINUE	CANT3300
	IF (LDIND .EQ. 2) GO TO 1000	CANT3310
	XYZDS(IDSPD1) = XYZDS(IDSPD1) + CDLXYZ(IDSPD1)	CANT3320
	XYZDS(IDSPD2) = XYZDS(IDSPD2) + CDLXYZ(IDSPD2)	CANT3330
	GO TO 1000	CANT3340
400	CONTINUE	CANT3350
	CALL ENERGY(FMVCTG(1,4))	CANT3360
	CALL OUTPUT	CANT3370
	IF (NITERS .LE. NITER) GO TO 405	CANT3380
	WRITE(6,403)	CANT3390
403	FORMAT (34H1MAXIMUM ITERATION NUMBER EXCEEDED)	CANT3400
	RETURN	CANT3410
405	CONTINUE	CANT3420
	IF (NDISPS .GE. NSTEP) RETURN	CANT3430
	IF(IFPATN .NE. 1) GO TO 430	CANT3440
427	CONTINUE	CANT3450
	IF (FMVCTG(IDSPDR,4) .GE. 0.0) GO TO 430	CANT3460
	WRITE(6,428)	CANT3470
428	FORMAT (21H1NEGATIVE NORMAL LOAD)	CANT3480
	RETURN	CANT3490
430	CONTINUE	CANT3500
	IF (LDIND .EQ. 2) GO TO 432	CANT3510
	XYZDS(IDSPD1) = 0.0	CANT3520
	XYZDS(IDSPD2) = 0.0	CANT3530
432	CONTINUE	CANT3540
	DO 434 I = 1, 3	CANT3550
434	CDLXYZ(I) = XYZDS(I)	CANT3560
	DO 436 I = 4, 15	CANT3570
436	CDLXYZ(I) = 0.0	CANT3580
	I1ST = 0	CANT3590
	IFIN = 0	CANT3600
	I2ND = 0	CANT3610
C	SAVE TOTAL MAIN STRUT LENGTH = STRUT 2 + STRUT 4	CANT3620
	STLNS = STLN(2) + STLN(4)	CANT3630
	NITERS = 1	CANT3640
C		CANT3650
C	SET STRUT SAVE VARIABLES FOR THIS POSITION	CANT3660
C		CANT3670
	ISTOP1 = 0	CANT3680
	DO 440 IST = 1, 4	CANT3690
	MNO = IMATL(IST)	CANT3700
	IF (INDPL(MNO) .NE. 2) GO TO 440	CANT3710
	CALL STRUT (IST, SPNGC(IST) , SPNGT(IST) , DISTC(IST) ,	CANT3720
1	DISTT(IST) , PFORC(IST) , PFORT(IST) ,	CANT3730
2	FOREVC(IST) , FOREVT(IST) , INDULC(IST) ,	CANT3740
3	INDULT(IST) , IPREV(IST) , STRP(IST) ,	CANT3750
4	IPOSC(IST) , IPOST(IST) , IRET(MNO) ,	CANT3760
5	SCHAX(MNO) , STMX(MNO) , CDC(1,MNO) ,	CANT3770
6	SRULT(MNO) , SRULC(MNO) , SRT(1,MNO) ,	CANT3780
7	SRC(1,MNO) , PFT(1,MNO) , PFC(1,MNO) ,	CANT3790
8	STR(IST) , AXIALF(IST) , AXSTIF(IST) ,	CANT3800
9	IFIRST(MNO) , NITEPS , CDT(1,MNO) ,	CANT3810
A	ISTOP2)	CANT3815
	IF (ISTOP2 .LT. 0) ISTOP1 = 1	CANT3820
440	CONTINUE	CANT3830

```
IF ( ISTOP1 .EQ. 1 ) RETURN  
GO TO 280  
END
```

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CANT3840  
CANT3850  
CANT3860
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SUPROUTINE BNDLDS
COMMON / CMAIN /
1  LGRTYP, LDIND , INITER, NPOINT, NREFP , NELEM , NUMB , BNDL 10
2  NSTEP , NITER , TOL , ENGABS, IFPATN, NFPCRL, FPCRLD(3), BNDL 20
3  IGLDIR, NMATL , IST , IMATL(4) , MNO , FPCRST(3), BNDL 30
4  INDPL(4) , ELASBN(4) , ELASAX(4) , BNDL 40
5  BINERT(4) , AREA(4) , IRET(4) , BNDL 50
6  IFIRST(4) , SCMAX(4) , STMAX(4) , BNDL 60
7  SRULC(4) , SRULT(4), CDC(5,4), SRC(5,4) , BNDL 70
8  CDT(5,4) , CRC(5,4) , SRT(5,4) , BNDL 80
9  PFC(5,4) , PFT(5,4) , NPN , XYZPOS(3,5) BNDL 90
9  , DC(3,3) , PHI , THTA , PSI BNDL 100
9  , CGYCDL(3), EGYCDL(3) BNDL 110
COMMON / CMAIN / BNDL 120
A  NITERS, NDISPS, STLN2 , BNDL 130
B  IDSPDR, IDSPD1, IDSPD2, SPDISP, COEFF , XYZDSP(3), BNDL 140
C  STLN(4) , STLNO(4) , STRP(4) , BNDL 150
D  SPNGC(4) , SPNGT(4) , DISTC(4) , BNDL 160
E  DISTT(4) , PFORC(4) , PFORT(4) , BNDL 170
F  FOREVC(4) , FOREVT(4) , IPOSC(4) , BNDL 180
G  IPOST(4) , INDULC(4) , INDULT(4) , BNDL 190
H  STR(4) , AXIALF(4) , AXSTIF(4) , BNDL 200
I  IPREV(4) , XYZDS(3) , SFORCE(30) , BNDL 210
J  CFORCE(30) , SUMFM4(6) , R24XYZ(3) , BNDL 220
K  TM(36) , FMVCTL(12,4) , FMVCTG(12,4) , BNDL 230
L  YSHRFP(4), ZSHRFP(4), YSHRMP(4), ZSHRMP(4), CGYXYZ(3), BNDL 240
M  YSHRFQ(4), ZSHRFQ(4), YSHRMQ(4), ZSHRMQ(4), EGYXYZ(3) BNDL 250
DOUBLE PRECISION A, B, C, D, E, F, AL24, AL45, AL25, AL, DELTA, BNDL 260
1  SINALF, COSALF, SINBET, COSBET, EB2, EB4, AI2, BNDL 270
2  AI4, TERM, TERM1, V2, V4, BM, RM1, RM3 BNDL 280
A=XYZPOS(1,5)-XYZPOS(1,2) BNDL 290
B=XYZPOS(2,5)-XYZPOS(2,2) BNDL 300
C=XYZPOS(3,5)-XYZPOS(3,2) BNDL 310
D=XYZPOS(1,5)-XYZPOS(1,4) BNDL 320
E=XYZPOS(2,5)-XYZPOS(2,4) BNDL 330
F=XYZPOS(3,5)-XYZPOS(3,4) BNDL 340
AL24 = DSQRT((A-D)**2 + (B-E)**2 + (C-F)**2) BNDL 350
AL45 = DSQRT( D*D + E*E + F*F ) BNDL 360
AL25 = DSQRT( A*A + B*B + C*C ) BNDL 370
AL = AL25 BNDL 380
DELTA = DSQRT((B*F-C*E)**2 + (C*D-A*F)**2 + (A*E-B*D)**2)/AL25 BNDL 390
SINALF = DELTA / AL24 BNDL 400
COSALF = DSQRT( 1. - SINALF*SINALF ) BNDL 410
SINBET = DELTA / AL45 BNDL 420
COSBET = DSQRT( 1. - SINBET * SINBET ) BNDL 430
AL24 = AL24 * COSALF BNDL 440
AL45 = AL45 * COSBET BNDL 450
IM2=IMATL(2) BNDL 460
IM4=IMATL(4) BNDL 470
EB2=ELASBN(IM2) BNDL 480
EB4=ELASBN(IM4) BNDL 490
AI2=BINERT(IM2) BNDL 500
AI4=BINERT(IM4) BNDL 510
TERM=(AL24**3*AL45/(2.*AL)-AL24**4*AL45/(3.*AL*AL))/(EB2*AI2)+ BNDL 520
1 (AL24**2*AL/3. - AL24**3 + AL24**4/AL - AL24**5/(3.*AL*AL))/ BNDL 530

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2	(EB4*AI4)	BNDL 560
	TERM1=DELTA/(TERM*AL)	BNDL 570
	V2=AL45*TERM1	BNDL 580
	V4=AL24*TERM1	BNDL 590
	BM=AL24*AL45*TERM1	BNDL 600
	YSHRFP(2) = V2	BNDL 610
	YSHRFQ(2) = -V2	BNDL 620
	YSHRFP(4) = V4	BNDL 630
	YSHRFQ(4) = -V4	BNDL 640
	ZSHRMQ(2) = BM	BNDL 650
	ZSHRMQ(4) = BM	BNDL 660
	RM1 = XYZPOS(1,4) + (C*C*D - A*C*F - A*B*E + B*B*D)/(AL25**2)	BNDL 670
	RM2 = XYZPOS(2,4) + (A*A*E - A*B*D - B*C*F + C*C*E)/(AL25**2)	BNDL 680
	RM3 = XYZPOS(3,4) + (B*B*F - B*C*E - A*C*D + A*A*F)/(AL25**2)	BNDL 690
	R24XYZ(1) = RM1	BNDL 700
	R24XYZ(2) = RM2	BNDL 710
	R24XYZ(3) = RM3	BNDL 720
	RETURN	BNDL 730
	END	BNDL 740

```

SUBROUTINE STFMCF( STIFML )
COMMON / CMAIN /
1  LGRTYP, LDIND , INITER, NPOINT, NREFP , NELEM , NUMB ,
2  NSTEP , NITER , TOL , ENGABS, IFPATN, NFPCRL, FPCRLD(3),
3  IGLDIR, NMATL , IST , IMATL(4) , MNO , FPCRST(3),
4  INDPL(4) , ELASBN(4) , ELASAX(4) ,
5  EINERT(4) , AREA(4) , IRET(4) ,
6  IFIRST(4) , SCMAX(4) , STMAX(4) ,
7  SRULC(4) , SRULT(4), CDC(5,4), SRC(5,4) ,
8  CDT(5,4) , CRC(5,4) , SRT(5,4) ,
9  PFC(3,4) , PFT(5,4) , NPN , XYZPOS(3,5)
9 , DC(3,3) , PHI , THTA , PSI
9 , CGYCDL(3), EGYCDL(3)
COMMON / CMAIN /
A  NITERS, NDISPS, STLN2 ,
R  IDSPDR, IDSPD1, IDSPD2, SPDISP, COEFF , XYZDSP(3),
C  STLN(4) , STLNO(4) , STRP(4) ,
D  SPNGC(4) , SPNGT(4) , DISTC(4) ,
E  DISTT(4) , PFORC (4) , PFCRT(4) ,
F  FOREVC(4) , FOREVT(4) , IPCSC(4) ,
G  IPOST(4) , INDULC(4) , INDULT(4) ,
H  STR(4) , AXIALF(4) , AXSTIF(4) ,
I  IPRLV(4) , XYZDS (3) , SFORCE(30) ,
J  CFORCE(30) , SUMFM4(6) , R24XYZ(3) ,
K  TR(9,4) , FMVCTL(12,4) , FMVCTG(12,4) ,
L  YSHRFP(4), ZSHRFP(4), YSHRMP(4), ZSHRMP(4), CGYXYZ(3),
M  YSHRFQ(4), ZSHRFQ(4), YSHRMQ(4), ZSHRMQ(4), EGYXYZ(3)
C
C  CALCULATE THE LOCAL STIFFNESS MATRIX FOR A BENDING STRUT
C
DIMENSION STIFML(12,12)
MNO = IMATL(IST)
DO 100 I = 2, 12
K = I - 1
DO 100 J = 1, K
100 STIFML(I,J) = 0.0
STIFML(1,1) = AXSTIF(IST)
STIFML(2,2) = 6.*AXIALF(IST)/(5.*STLN(IST)) +
1 12.*ELASBN(MNO)*BINERT(MNO)/ STLN(IST)**3
STIFML(3,3) = STIFML(2,2)
STIFML(4,4) = STIFML(1,1)
STIFML(5,5) = STIFML(2,2)
STIFML(6,6) = STIFML(2,2)
STIFML(7,7) = 0.0
STIFML(8,8) = 2.*AXIALF(IST)*STLN(IST)/15.+4.*ELASBN(MNO)
1 *BINERT(MNO) / STLN(IST)
STIFML(9,9) = STIFML(8,8)
STIFML(10,10) = 0.0
STIFML(11,11) = STIFML(8,8)
STIFML(12,12) = STIFML(8,8)
STIFML(4,1) = -STIFML(1,1)
STIFML(5,2) = -STIFML(2,2)
STIFML(6,3) = -STIFML(2,2)
STIFML(8,3) = -AXIALF(IST)*.1 - 6.*ELASBN(MNO)*BINERT(MNO)
1 / STLN(IST)**2

```

STIFML(8,6) = -STIFML(8,3)	STBN 560
STIFML(9,2) = -STIFML(8,3)	STBN 570
STIFML(9,5) = STIFML(8,3)	STBN 580
STIFML(11,3) = STIFML(8,3)	STBN 590
STIFML(11,6) = -STIFML(8,3)	STBN 600
STIFML(12,2) = -STIFML(8,3)	STBN 610
STIFML(12,5) = STIFML(8,3)	STBN 620
STIFML(11,8) = -AXIALF(IST)*STLN(IST)/30.	STBN 630
1 + 2.*ELASBN(MNO)*BINERT(MNO)/STLN(IST)	STBN 640
STIFML(12,9) = STIFML(11,8)	STBN 650
DO 200 I = 2, 12	STBN 660
K = I - 1	STBN 670
DO 200 J = 1, K	STBN 680
200 STIFML(J,I) = STIFML(I,J)	STBN 690
RETURN	STBN 700
END	STBN 710

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APPENDIX I

PROGRAM LISTING

LANDING LOADS AND MOTIONS PROGRAM

OVERLAY (LLMPT5, 0, 0)

OVER 10

```
C
C PROGRAM LLMP (INPUT,OUTPUT,TAPE5=INPUT,TAPE6=OUTPUT,TAPE3,TAPE4) LLMP 10
C LANDING LOADS AND MOTIONS PROGRAM LLMP 20
C MASTER AGREEMENT, CONTRACT NAS1-8137, TASK ORDER NUMBER FIVE LLMP 30
C MCDONNELL DOUGLAS ASTRONAUTICS COMPANY - EAST LLMP 40
C LLMP 50
C LLMP 60
C FORTRAN I/O UNIT 3 - SECONDARY TIME HISTORY DATA PLACED ON LLMP 70
C THIS FILE (NFORC=0, NO SECONDARY TIME LLMP 80
C HISTORY OUTPUT - NFORC=1, SECONDARY LLMP 90
C HISTORY OUTPUT ON TAPE3). LLMP 100
C FORTRAN I/O UNIT 4 - WORK FILE ASSOCIATED WITH MULTIPLE DATA LLMP 110
C CASES. LLMP 120
C FORTRAN I/O UNIT 5 - CARD INPUT DATA. LLMP 130
C FORTRAN I/O UNIT 6 - PRINTED OUTPUT. LLMP 140
C LLMP 150
C COMMON COMINT(1400) LLMP 160
C EQUIVALENCE ( COMINT ( 362 ), IDSETN ) LLMP 170
C EQUIVALENCE ( COMINT (1399), NOCASE ) LLMP 180
C EQUIVALENCE ( COMINT (1400), TYMIN ) LLMP 190
C COMMON / THISV / TIMHSA(275) LLMP 200
C DO 1000 I = 1, 1400 LLMP 210
1000 COMINT(I) = 0.0 LLMP 220
C DO 2000 I = 1, 275 LLMP 230
2000 TIMHSA(I) = 0.0 LLMP 240
C LLMP 250
C CALL THE INPUT AND INITIALIZATION ROUTINES LLMP 260
C LLMP 270
C 1 CONTINUE LLMP 280
C LLMP 290
C CALL SECOND(TYMIN) LLMP 300
C LLMP 310
C CALL OVERLAY(6LLLMP5,1,0,6HRECALL ) LLMP 320
C LLMP 330
C CALL THE LANDER LANDING, LOADS, AND MOTION SIMULATION LLMP 340
C ROUTINES LLMP 350
C CALL OVERLAY(6LLLMP5,2,0,6HRECALL ) LLMP 360
C LLMP 370
C CALL SECOND(TYMOUT) LLMP 380
C LLMP 390
C RUNTYM=TYMOUT-TYMIN LLMP 400
C WRITE(6,3000)NOCASE,RUNTYM LLMP 410
C GO TO 1 LLMP 420
C LLMP 430
C LLMP 440
C FORMAT STATEMENT LLMP 450
3000 FORMAT(/10X,8HCASE NO ,I10,9H RAN FOR ,F7.3,11H CP SECONDS) LLMP 460
C END LLMP 470
```

OVERLAY (LLMPT5, 1, 0)

OVER 20

PROGRAM READIT

READ 10

THIS PORTION OF THE PROGRAM READS THE INPUT DATA FROM SEQUENCED DATA CARDS.

READ 20

READ 30

READ 40

READ 50

THE CARDS MUST BE PREPARED IN THE FOLLOWING FORMAT

READ 60

READ 70

COLUMN	1-4	6-9	11-20	21-30	31-40	41-50	51-60
	ICNTRL	CARD	DATA	DATA	DATA	DATA	DATA
		SEQ.					
		RT. JUST.					

READ 80

READ 90

READ 100

READ 110

READ 120

DIMENSION GC(4,5), GCD(4,5), GCDD(4,5)

READ 130

READ 140

DIMENSION XFPS(4,5), YFPS(4,5), ZFPS(4,5),

READ 150

1 XFPSD(4,5), YFPSD(4,5), ZFPSD(4,5),

READ 160

2 XFPSDD(4,5), YFPSDD(4,5), ZFPSDD(4,5)

READ 170

READ 180

DIMENSION DUMMY(5),KONT(50),JKONT(20),NOCARD(500)

READ 190

READ 200

COMMON / THISV/ TIMHSA(1), ATTH(5),AM(6,6),AME1(3),INDFPI(6),

READ 210

1 INDFPC(6),

READ 220

2 STRPDS(10), STRPMS(5), IPOCDs(10), IPOCMS(5), URCDS (10),

READ 230

3 URTDS (10), URCMS (5), URTMS (5), SETCDS(10), SETTDS(10),

READ 240

4 SETCMS(5), SETTMS(5), INDCDS(10), INDTDS(10), INDCMS(10),

READ 250

5 INDTMS(5), PRFCDS (10), PRFTDS (10), PRFCMS (5), PRFTMS (5),

READ 260

6 IPRDS (10), IPRMS (5), FRVDSC(10), FRVDST(10), FRVMSC(5),

READ 270

L FRVMST(5), IPOTDS(10), IPOTMS(5)

READ 280

EQUIVALENCE (NFTPDS, NOLEG)

READ 290

COMMON COMINT(400)

READ 300

EQUIVALENCE (COMINT(1), ISAVCM)

READ 310

EQUIVALENCE (COMINT(2), JTEST)

READ 320

EQUIVALENCE (COMINT(3), IBOTM)

READ 330

EQUIVALENCE (COMINT(4), XSD)

READ 340

EQUIVALENCE (COMINT(8), XSDD)

READ 350

EQUIVALENCE (COMINT(12), YS)

READ 360

EQUIVALENCE (COMINT(16), YSD)

READ 370

EQUIVALENCE (COMINT(20), YSDD)

READ 380

EQUIVALENCE (COMINT(24), ZS)

READ 390

EQUIVALENCE (COMINT(28), ZSD)

READ 400

EQUIVALENCE (COMINT(32), ZSDD)

READ 410

EQUIVALENCE (COMINT(36), PHI)

READ 420

EQUIVALENCE (COMINT(40), PHID)

READ 430

EQUIVALENCE (COMINT(44), WX)

READ 440

EQUIVALENCE (COMINT(48), WXD)

READ 450

EQUIVALENCE (COMINT(52), THTA)

READ 460

EQUIVALENCE (COMINT(56), THTAD)

READ 470

EQUIVALENCE (COMINT(60), WY)

READ 480

EQUIVALENCE (COMINT(64), WYD)

READ 490

EQUIVALENCE (COMINT(68), PSI)

READ 500

EQUIVALENCE (COMINT(72), PSID)

READ 510

EQUIVALENCE (COMINT(76), WZ)

READ 520

EQUIVALENCE (COMINT(80), WZD)	READ 530
EQUIVALENCE (COMINT(84), GC)	READ 540
EQUIVALENCE (COMINT(104), GCD)	READ 550
EQUIVALENCE (COMINT(124), GCDD)	READ 560
EQUIVALENCE (COMINT(144), XFPS)	READ 570
EQUIVALENCE (COMINT(164), XFPSD)	READ 580
EQUIVALENCE (COMINT(184), XFPSDD)	READ 590
EQUIVALENCE (COMINT(204), YFPS)	READ 600
EQUIVALENCE (COMINT(264), ZFPS)	READ 610
EQUIVALENCE (COMINT(224), YFPSD)	READ 620
EQUIVALENCE (COMINT(244), YFPSDD)	READ 630
EQUIVALENCE (COMINT(284), ZFPSD)	READ 640
EQUIVALENCE (COMINT(304), ZFPSDD)	READ 650
EQUIVALENCE (COMINT(324), TIME)	READ 660
EQUIVALENCE (COMINT(325), HMAX)	READ 670
EQUIVALENCE (COMINT(326), HMIN)	READ 680
EQUIVALENCE (COMINT(327), EMIN)	READ 690
EQUIVALENCE (COMINT(328), EMAX)	READ 700
EQUIVALENCE (COMINT(329), XSI)	READ 710
EQUIVALENCE (COMINT (337), HZ)	READ 720
EQUIVALENCE (COMINT(338), CUTERR)	READ 730
EQUIVALENCE (COMINT(339), IP)	READ 740
EQUIVALENCE (COMINT(340), IVARH)	READ 750
EQUIVALENCE (COMINT(341), IMTH)	READ 760
EQUIVALENCE (COMINT(342), IPRNT)	READ 770
EQUIVALENCE (COMINT(343), IFIN)	READ 780
EQUIVALENCE (COMINT(344), IAD)	READ 790
EQUIVALENCE (COMINT(352), IND)	READ 800
EQUIVALENCE (COMINT(360), JCUT)	READ 810
EQUIVALENCE (COMINT(361), IPTCNT)	READ 820
EQUIVALENCE (COMINT (362), IDSETN)	READ 830
EQUIVALENCE (COMINT(363), IVAL)	READ 840
EQUIVALENCE (COMINT(364), XS)	READ 850
COMINT(364-367) USED BY XS	READ 860
EQUIVALENCE (COMINT (1399), NOCASE)	READ 870
COMMON	READ 880
1 CBMASS, CBIXX , CBIXZ , CBIYY , CBIYZ , CBIZZ , FPMASS,CBIXY,	READ 890
2 DC(3,3) , XFP(5), YFP(5), ZFP(5), WNX(5), WNY(5),	READ 900
3 WNZ(5), PX(5) , PY(5) , PZ(5) , GM(5) , OMEGA(5) ,	READ 910
4 GRAV , GRAVE , ZETA , FTS (6) , FSXSI(5) ,	READ 920
5 FSYSI(5) , FSZSI(5) , SOILX(5) ,	READ 930
6 SOILY(5) , SOILZ(5) , PMSX(5,5) ,	READ 940
7 PMSY(5,5) , PMSZ(5,5) , PDSX(10,5) ,	READ 950
8 PDSY(10,5) , PDSZ(10,5) , FLXS , FLYS , FLZS ,	READ 960
9 TLXL , TLYL , TLZL , SLO , XMSCB(5) ,	READ 970
C YMSCB(5) , ZMSCB(5) , XDSCB(10) ,	READ 980
D YDSCB(10) , ZDSCB(10) , ILEG , IMS ,	READ 990
E FSTX , FSTY , FSTZ , PVCBX , PVCBY , PVCBZ ,	READ1000
F PVFPX , PVFPY , PVFPZ , NOLEG , SLOMS(5) ,	READ1010
G SLODS(10)	READ1020
COMMON	READ1030
1 PFCMS(5) , PFCDS(5) , PFTDS(5) ,	READ1040
2 PFTMS(5) , SRCDS(5) , SRCMS(5) ,	READ1050
3 SRTDS(5) , SRTMS(5) , COEFDS, COEFMS, GAMDS ,	READ1060
4 GAMMS , SRUCDS, SRUCMS, SRUTDS, SRUTMS, SCMXDS, SCMXMS,	READ1070
5 STMXDS, STMXMS, CDCDS(5) , CDCMS(5) ,	READ1080

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6   CDTDS(5)      , CDTMS(5)      , FRICDS, FRICMS, IRETDS,      READ1090
7   IRETMS, STROKE(10)      , STRKDS(10)      , STRKMS( 5)      ,      READ1100
8   LENGTH,CDXS,CDYS,CDZS , NTYPE , RAD(4) , SS(4) , ATTHCK(3) ,      READ1110
9   ATTPRS(3)      , ADIST , SOILP(3)      , NMODES,      READ1120
A   COEF , GAMMA , FRIC , SCMAX , STMAX , CDC(5), CDT(5),      READ1130
B   SRULT , SRULC , SRT(5), SRC(5), PFT(5), PFC(5), GFLEGS(5),      READ1140
C   CURMSV, CURMSL, INDFXD, INDFYD, INDFZD, INDFXR, INDFYR,      READ1150
D   INDFZR, TIMAX , DRAGST, IFP      READ1160
C
COMMON
1CMS,CDCONT,SOILNU,SLRHO,      NOOUT,XOUT(10),YOUT(10),ZOUT(10),      READ1180
2MODEIN,POUTX(10,5),POUTY(10,5),POUTZ(10,5),PCGX(5),PCGY(5),      READ1190
3PCGZ(5),AEI1,AEI2,FORMS(5),FORDS(10),FORCE,NFORC,SAVMSX(5),      READ1200
4SAVMSZ(5),SAVDSX(10),SAVDSY(10),SAVDSZ(10),IQUOUT,GSINZT,      READ1210
5GCOSZT,STAB,STABVL,ISTAB,JCKSAB,VELX,VELY,VELZ,SAVMSY(5)      READ1220
COMMON
1   SMXMSC(5), TMXMSC(5), SMXMST(5), TMXMST(5),      READ1230
2   SMXDSC(10), TMXDSC(10), SMXDST(10), TMXDST(10)      READ1240
3   ,SLNGMS(5) ,SLNGDS(10), CURDSL, INLEG , IFPRT,      READ1250
4   IMPACT(5) ,IPRTFP(5), KOUNT(5) ,ANGX, ANGY, ANGZ      READ1260
INTEGER STOP      READ1270
DATA NEXT/4HNEXT/,STOP/4HSTOP/,NOGO/0/      READ1280
IF ( ISAVCM .EQ. 0 ) GO TO 4000      READ1290
C
C   INITIALIZE COMMONS BEFORE READING NEXT DATA CASE      READ1300
C
REWIND 4      READ1310
READ (4) (COMINT(I),I=1,1399),(TIMHSA(I),I=1,275)      READ1320
4000 CONTINUE      READ1330
DO 5000 I=1,50      READ1340
5000 KONT(I)=0      READ1350
DO 5001 I=1,20      READ1360
5001 JKONT(I)=0      READ1370
MMM=0      READ1380
WRITE(6,9000)      READ1390
9000 FORMAT(1H1,43X,      READ1400
*49HLANDING LOADS AND MOTIONS PROGRAM - LEGGED LANDER/38X,      READ1410
*61HMASTER AGREEMENT, CONTRACT NAS1-8137, TASK ORDER NUMBER FIVE      READ1420
*/46X,      READ1430
*45HMC DONNELL DOUGLAS ASTRONAUTICS COMPANY - EAST ///63X,      READ1440
*10HINPUT DATA ////)      READ1450
9999 READ(5,9001)ICNTRL,NCARD,(DUMMY(I),I=1,5)      READ1460
9001 FORMAT(A4,1X,I4,1X,5E10.3)      READ1470
IF(ICNTRL.EQ.NEXT) GO TO 7999      READ1480
IF(ICNTRL.EQ.STOP) STOP      READ1490
MMM=MMM+1      READ1500
NOCARD(MMM)=NCARD      READ1510
IF(NCARD.LE.0)GO TO 9004      READ1520
IF(NCARD.GT.42)GO TO 9003      READ1530
GO TO (1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,      READ1540
*      22,23,24,25,26,27,28,29,30,31,32,33,34,35,36,37,38,39,      READ1550
*      40,41,42      READ1560
*      ),NCARD      READ1570
9003 CONTINUE      READ1580
JJJ=NCARD/100      READ1590
IF(JJJ.LE.0.OR.JJJ.GT.17)GO TO 9004      READ1600
      READ1610
      READ1620
      READ1630
      READ1640

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	GO TO (100,200,300,400,500,600,700,800,900,1000,1100,1200,	READ1650
	* 1300,1400,1500,1600,1700	READ1660
	*),JJJ	READ1670
9004	CONTINUE	READ1680
	WRITE(6,9002) NCARD	READ1690
9002	FORMAT(24H INVALID CARD NUMBER -- ,I4)	READ1700
	NOGO=1	READ1710
	GO TO 9999	READ1720
C		READ1730
C	ASSIGNMENT OF INPUT DATA	READ1740
C		READ1750
1	CONTINUE	READ1760
	KONT(NCARD)=KONT(NCARD)+1	READ1770
	NOCASE =DUMMY(1)	READ1780
	GO TO 9999	READ1790
C		READ1800
2	CONTINUE	READ1810
	KONT(NCARD)=KONT(NCARD)+1	READ1820
	TIMAX =DUMMY(1)	READ1830
	IPTCNT =DUMMY(2)	READ1840
	INLEG =DUMMY(3)	READ1850
	IFPRT =DUMMY(4)	READ1860
	GO TO 9999	READ1870
C		READ1880
3	CONTINUE	READ1890
	KONT(NCARD)=KONT(NCARD)+1	READ1900
	NMODES =DUMMY(1)	READ1910
	NOOUT =DUMMY(2)	READ1920
	GO TO 9999	READ1930
C		READ1940
4	CONTINUE	READ1950
	KONT(NCARD)=KONT(NCARD)+1	READ1960
	INDFXD =DUMMY(1)	READ1970
	INDFYD =DUMMY(2)	READ1980
	INDFZD =DUMMY(3)	READ1990
	GO TO 9999	READ2000
C		READ2010
5	CONTINUE	READ2020
	KONT(NCARD)=KONT(NCARD)+1	READ2030
	INDFXR =DUMMY(1)	READ2040
	INDFYR =DUMMY(2)	READ2050
	INDFZR =DUMMY(3)	READ2060
	GO TO 9999	READ2070
C		READ2080
6	CONTINUE	READ2090
	KONT(NCARD)=KONT(NCARD)+1	READ2100
	HMAX =DUMMY(1)	READ2110
	HMIN =DUMMY(2)	READ2120
	EMAX =DUMMY(3)	READ2130
	EMIN =DUMMY(4)	READ2140
	IP =DUMMY(5)	READ2150
	GO TO 9999	READ2160
C		READ2170
7	CONTINUE	READ2180
	KONT(NCARD)=KONT(NCARD)+1	READ2190
	IVARH =DUMMY(1)	READ2200

	IMTH	=DUMMY(2)	READ2210
	CUTERR	=DUMMY(3)	READ2220
	GO TO 9999		READ2230
C			READ2240
	8 CONTINUE		READ2250
	KONT(NCARD)=KONT(NCARD)+1		READ2260
	NFORC	=DUMMY(1)	READ2270
	IQUOUT	=DUMMY(2)	READ2280
	JCKSAB	=DUMMY(3)	READ2290
	IDSETN	=DUMMY(4)	READ2300
	GO TO 9999		READ2310
C			READ2320
	9 CONTINUE		READ2330
	KONT(NCARD)=KONT(NCARD)+1		READ2340
	ZETA	=DUMMY(1)	READ2350
	GRAV	=DUMMY(2)	READ2360
	GRAVE	=DUMMY(3)	READ2370
	GO TO 9999		READ2380
C			READ2390
	10 CONTINUE		READ2400
	KONT(NCARD)=KONT(NCARD)+1		READ2410
	ANGX	=DUMMY(1)	READ2420
	ANGY	=DUMMY(2)	READ2430
	ANGZ	=DUMMY(3)	READ2440
	GO TO 9999		READ2450
C			READ2460
	11 CONTINUE		READ2470
	KONT(NCARD)=KONT(NCARD)+1		READ2480
	WX	=DUMMY(1)	READ2490
	WY	=DUMMY(2)	READ2500
	WZ	=DUMMY(3)	READ2510
	GO TO 9999		READ2520
C			READ2530
	12 CONTINUE		READ2540
	KONT(NCARD)=KONT(NCARD)+1		READ2550
	VELX	=DUMMY(1)	READ2560
	VELY	=DUMMY(2)	READ2570
	VELZ	=DUMMY(3)	READ2580
	GO TO 9999		READ2590
C			READ2600
	13 CONTINUE		READ2610
	KONT(NCARD)=KONT(NCARD)+1		READ2620
	CBMASS	=DUMMY(1)	READ2630
	GO TO 9999		READ2640
C			READ2650
	14 CONTINUE		READ2660
	KONT(NCARD)=KONT(NCARD)+1		READ2670
	CBIXX	=DUMMY(1)	READ2680
	CBIYY	=DUMMY(2)	READ2690
	CBIZZ	=DUMMY(3)	READ2700
	GO TO 9999		READ2710
C			READ2720
	15 CONTINUE		READ2730
	KONT(NCARD)=KONT(NCARD)+1		READ2740
	CBIXY	=DUMMY(1)	READ2750
	CBIXZ	=DUMMY(2)	READ2760

	CBIYZ	=DUMMY(3)	READ2770
	GO TO 9999		READ2780
C			READ2790
	100 CONTINUE		READ2800
	JKONT(JJJ)=JKONT(JJJ)+1		READ2810
	II=JKONT(JJJ)		READ2820
	XOUT(II)	=DUMMY(1)	READ2830
	YOUT(II)	=DUMMY(2)	READ2840
	ZOUT(II)	=DUMMY(3)	READ2850
	GO TO 9999		READ2860
C			READ2870
	16 CONTINUE		READ2880
	KONT(NCARD)=KONT(NCARD)+1		READ2890
	FPMASS	=DUMMY(1)	READ2900
	GO TO 9999		READ2910
C			READ2920
	17 CONTINUE		READ2930
	KONT(NCARD)=KONT(NCARD)+1		READ2940
	DO 7000 I=1,4		READ2950
7000	RAD(I)	=DUMMY(I)	READ2960
	GO TO 9999		READ2970
C			READ2980
	18 CONTINUE		READ2990
	KONT(NCARD)=KONT(NCARD)+1		READ3000
	DO 7001 I=1,4		READ3010
7001	SS(I)	=DUMMY(I)	READ3020
	GO TO 9999		READ3030
C			READ3040
	19 CONTINUE		READ3050
	KONT(NCARD)=KONT(NCARD)+1		READ3060
	DO 7002 I=1,3		READ3070
7002	ATTHCK(I)	=DUMMY(I)	READ3080
	GO TO 9999		READ3090
C			READ3100
	20 CONTINUE		READ3110
	KONT(NCARD)=KONT(NCARD)+1		READ3120
	DO 7003 I=1,3		READ3130
7003	ATTPRS(I)	=DUMMY(I)	READ3140
	GO TO 9999		READ3150
C			READ3160
	21 CONTINUE		READ3170
	KONT(NCARD)=KONT(NCARD)+1		READ3180
	NTYPE	=DUMMY(1)	READ3190
	GO TO 9999		READ3200
C			READ3210
	22 CONTINUE		READ3220
	KONT(NCARD)=KONT(NCARD)+1		READ3230
	DO 7004 I=1,3		READ3240
7004	SOILP(I)	=DUMMY(I)	READ3250
	GO TO 9999		READ3260
C			READ3270
	23 CONTINUE		READ3280
	KONT(NCARD)=KONT(NCARD)+1		READ3290
	NOLEG	=DUMMY(1)	READ3300
	ILEG	=DUMMY(2)	READ3310
	DRAGST	=DUMMY(3)	READ3320

	GO TO 9999	READ3330
C		READ3340
200	CONTINUE	READ3350
	JKONT(JJJ)=JKONT(JJJ)+1	READ3360
	II=JKONT(JJJ)	READ3370
	XFP(II) =DUMMY(1)	READ3380
	YFP(II) =DUMMY(2)	READ3390
	ZFP(II) =DUMMY(3)	READ3400
	GO TO 9999	READ3410
C		READ3420
300	CONTINUE	READ3430
	JKONT(JJJ)=JKONT(JJJ)+1	READ3440
	II=JKONT(JJJ)	READ3450
	XMSCB(II) =DUMMY(1)	READ3460
	YMSCB(II) =DUMMY(2)	READ3470
	ZMSCB(II) =DUMMY(3)	READ3480
	GO TO 9999	READ3490
C		READ3500
24	CONTINUE	READ3510
	KONT(NCARD)=KONT(NCARD)+1	READ3520
	DO 7005 I=1,5	READ3530
7005	PFCMS(I) =DUMMY(I)	READ3540
	GO TO 9999	READ3550
C		READ3560
25	CONTINUE	READ3570
	KONT(NCARD)=KONT(NCARD)+1	READ3580
	DO 7006 I=1,5	READ3590
7006	PFTMS(I) =DUMMY(I)	READ3600
	GO TO 9999	READ3610
C		READ3620
26	CONTINUE	READ3630
	KONT(NCARD)=KONT(NCARD)+1	READ3640
	DO 7007 I=1,5	READ3650
7007	CDCMS(I) =DUMMY(I)	READ3660
	GO TO 9999	READ3670
C		READ3680
27	CONTINUE	READ3690
	KONT(NCARD)=KONT(NCARD)+1	READ3700
	DO 7008 I=1,5	READ3710
7008	CDTMS(I) =DUMMY(I)	READ3720
	GO TO 9999	READ3730
C		READ3740
28	CONTINUE	READ3750
	KONT(NCARD)=KONT(NCARD)+1	READ3760
	DO 7009 I=1,5	READ3770
7009	SRCMS(I) =DUMMY(I)	READ3780
	GO TO 9999	READ3790
C		READ3800
29	CONTINUE	READ3810
	KONT(NCARD)=KONT(NCARD)+1	READ3820
	DO 7010 I=1,5	READ3830
7010	SRTMS(I) =DUMMY(I)	READ3840
	GO TO 9999	READ3850
C		READ3860
30	CONTINUE	READ3870
	KONT(NCARD)=KONT(NCARD)+1	READ3880

	SCMXMS	=DUMMY(1)	READ3890
	STMXMS	=DUMMY(2)	READ3900
	SRUCMS	=DUMMY(3)	READ3910
	SRUTMS	=DUMMY(4)	READ3920
	GO TO 9999		READ3930
C	31 CONTINUE		READ3940
	KONT(NCARD)=KONT(NCARD)+1		READ3950
	IRETMS	=DUMMY(1)	READ3960
	GO TO 9999		READ3970
C	32 CONTINUE		READ3980
	KONT(NCARD)=KONT(NCARD)+1		READ3990
	FRICMS	=DUMMY(1)	READ4000
	COEFMS	=DUMMY(2)	READ4010
	GAMMS	=DUMMY(3)	READ4020
	AEI1	=DUMMY(4)	READ4030
	AEI2	=DUMMY(5)	READ4040
	GO TO 9999		READ4050
C	400 CONTINUE		READ4060
	JKONT(JJJ)=JKONT(JJJ)+1		READ4070
	II=JKONT(JJJ)		READ4080
	XDSCB(II)	=DUMMY(1)	READ4090
	YDSCB(II)	=DUMMY(2)	READ4100
	ZDSCB(II)	=DUMMY(3)	READ4110
	GO TO 9999		READ4120
C	33 CONTINUE		READ4130
	KONT(NCARD)=KONT(NCARD)+1		READ4140
	DO 7011 I=1,5		READ4150
7011	PFCDS(I)	=DUMMY(I)	READ4160
	GO TO 9999		READ4170
C	34 CONTINUE		READ4180
	KONT(NCARD)=KONT(NCARD)+1		READ4190
	DO 7012 I=1,5		READ4200
7012	PFTDS(I)	=DUMMY(I)	READ4210
	GO TO 9999		READ4220
C	35 CONTINUE		READ4230
	KONT(NCARD)=KONT(NCARD)+1		READ4240
	DO 7013 I=1,5		READ4250
7013	CDCDS(I)	=DUMMY(I)	READ4260
	GO TO 9999		READ4270
C	36 CONTINUE		READ4280
	KONT(NCARD)=KONT(NCARD)+1		READ4290
	DO 7014 I=1,5		READ4300
7014	CDTDS(I)	=DUMMY(I)	READ4310
	GO TO 9999		READ4320
C	37 CONTINUE		READ4330
	KONT(NCARD)=KONT(NCARD)+1		READ4340
	DO 7015 I=1,5		READ4350
7015	SRCDS(I)	=DUMMY(I)	READ4360
			READ4370
			READ4380
			READ4390
			READ4400
			READ4410
			READ4420
			READ4430
			READ4440

	GO TO 9999	READ4450
C		READ4460
	38 CONTINUE	READ4470
	KONT(NCARD)=KONT(NCARD)+1	READ4480
	DO 7016 I=1,5	READ4490
	7016 SRTDS(I) =DUMMY(I)	READ4500
	GO TO 9999	READ4510
C		READ4520
	39 CONTINUE	READ4530
	KONT(NCARD)=KONT(NCARD)+1	READ4540
	SCMXDS =DUMMY(1)	READ4550
	STMXDS =DUMMY(2)	READ4560
	SRUCDS =DUMMY(3)	READ4570
	SRUTDS =DUMMY(4)	READ4580
	GO TO 9999	READ4590
C		READ4600
	40 CONTINUE	READ4610
	KONT(NCARD)=KONT(NCARD)+1	READ4620
	IRETDS =DUMMY(1)	READ4630
	GO TO 9999	READ4640
C		READ4650
	41 CONTINUE	READ4660
	KONT(NCARD)=KONT(NCARD)+1	READ4670
	FRICDS =DUMMY(1)	READ4680
	COEFDS =DUMMY(2)	READ4690
	GAMDS =DUMMY(3)	READ4700
	GO TO 9999	READ4710
C		READ4720
	42 CONTINUE	READ4730
	KONT(NCARD)=KONT(NCARD)+1	READ4740
	MODEIN =DUMMY(1)	READ4750
	GO TO 9999	READ4760
C		READ4770
	500 CONTINUE	READ4780
	JKONT(JJJ)=JKONT(JJJ)+1	READ4790
	II=JKONT(JJJ)	READ4800
	GM(II) =DUMMY(1)	READ4810
	OMEGA(II) =DUMMY(2)	READ4820
	GO TO 9999	READ4830
C		READ4840
	600 CONTINUE	READ4850
	JKONT(JJJ)=JKONT(JJJ)+1	READ4860
	II=JKONT(JJJ)	READ4870
	WNX(II) =DUMMY(1)	READ4880
	WNY(II) =DUMMY(2)	READ4890
	WNZ(II) =DUMMY(3)	READ4900
	GO TO 9999	READ4910
C		READ4920
	700 CONTINUE	READ4930
	JKONT(JJJ)=JKONT(JJJ)+1	READ4940
	II=JKONT(JJJ)	READ4950
	PX(II) =DUMMY(1)	READ4960
	PY(II) =DUMMY(2)	READ4970
	PZ(II) =DUMMY(3)	READ4980
	GO TO 9999	READ4990
C		READ5000

800	CONTINUE		READ5010
	JKONT(JJJ)=JKONT(JJJ)+1		READ5020
	II=JKONT(JJJ)		READ5030
	DO 7017 I=1,5		READ5040
7017	PMSX(II,I) =DUMMY(I)		READ5050
	GO TO 9999		READ5060
C			READ5070
			READ5080
900	CONTINUE		READ5090
	JKONT(JJJ)=JKONT(JJJ)+1		READ5100
	II=JKONT(JJJ)		READ5110
	DO 7018 I=1,5		READ5120
7018	PMSY(II,I) =DUMMY(I)		READ5130
	GO TO 9999		READ5140
C			READ5150
			READ5160
1000	CONTINUE		READ5170
	JKONT(JJJ)=JKONT(JJJ)+1		READ5180
	II=JKONT(JJJ)		READ5190
	DO 7019 I=1,5		READ5200
7019	PMSZ(II,I) =DUMMY(I)		READ5210
	GO TO 9999		READ5220
C			READ5230
			READ5240
1100	CONTINUE		READ5250
	JKONT(JJJ)=JKONT(JJJ)+1		READ5260
	II=JKONT(JJJ)		READ5270
	DO 7020 I=1,5		READ5280
7020	PDSX(II,I) =DUMMY(I)		READ5290
	GO TO 9999		READ5300
C			READ5310
			READ5320
1200	CONTINUE		READ5330
	JKONT(JJJ)=JKONT(JJJ)+1		READ5340
	II=JKONT(JJJ)		READ5350
	DO 7021 I=1,5		READ5360
7021	PDSY(II,I) =DUMMY(I)		READ5370
	GO TO 9999		READ5380
C			READ5390
			READ5400
1300	CONTINUE		READ5410
	JKONT(JJJ)=JKONT(JJJ)+1		READ5420
	II=JKONT(JJJ)		READ5430
	DO 7022 I=1,5		READ5440
7022	PDSZ(II,I) =DUMMY(I)		READ5450
	GO TO 9999		READ5460
C			READ5470
			READ5480
1400	CONTINUE		READ5490
	JKONT(JJJ)=JKONT(JJJ)+1		READ5500
	II=JKONT(JJJ)		READ5510
	DO 7023 I=1,5		READ5520
7023	POUTX(II,I) =DUMMY(I)		READ5530
	GO TO 9999		READ5540
C			READ5550
			READ5560
1500	CONTINUE		
	JKONT(JJJ)=JKONT(JJJ)+1		
	II=JKONT(JJJ)		
	DO 7024 I=1,5		
7024	POUTY(II,I) =DUMMY(I)		
	GO TO 9999		
C			

1600	CONTINUE	READ5570
	JKONT(JJJ)=JKONT(JJJ)+1	READ5580
	II=JKONT(JJJ)	READ5590
	DO 7025 I=1,5	READ5600
7025	POUTZ(II,I) =DUMMY(I)	READ5610
	GO TO 9999	READ5620
C		READ5630
1700	CONTINUE	READ5640
	JKONT(JJJ)=JKONT(JJJ)+1	READ5650
	II=JKONT(JJJ)	READ5660
	PCGX(II) =DUMMY(1)	READ5670
	PCGY(II) =DUMMY(2)	READ5680
	PCGZ(II) =DUMMY(3)	READ5690
	GO TO 9999	READ5700
C		READ5710
C	CHECK INPUT DATA	READ5720
C		READ5730
7999	CONTINUE	READ5740
	DO 8000 I=1,42	READ5750
	IF(KONT(I).EQ.1) GO TO 8000	READ5760
	IF(KONT(I).EQ.0.AND.JTEST.EQ.1) GO TO 8000	READ5770
	WRITE(6,8001)I	READ5780
	NOGO=1	READ5790
8000	CONTINUE	READ5800
8001	FORMAT(/45H INCORRECT NUMBER OF DATA CARDS - CARD NUMBER,I5)	READ5810
	DO 8007 I=1,17	READ5820
	II=I*100	READ5830
	IF(JKONT(I).EQ.0.AND.JTEST.EQ.1) GO TO 8007	READ5840
	GO TO (8002,8003,8003,8004,8005,8005,8005,8006,8006, 8006,8012,	READ5850
	* 8012,8012,8013,8013,8013,8005),I	READ5860
8002	IF(JKONT(I).EQ.NOOUT) GO TO 8007	READ5870
	NOGO=1	READ5880
	WRITE(6,8008)II	READ5890
	GO TO 8007	READ5900
8003	IF(JKONT(I).EQ.NOLEG) GO TO 8007	READ5910
	NOGO=1	READ5920
	WRITE(6,8008)II	READ5930
	GO TO 8007	READ5940
8004	IF(JKONT(I).EQ.2*NOLEG) GO TO 8007	READ5950
	NOGO=1	READ5960
	WRITE(6,8008)II	READ5970
	GO TO 8007	READ5980
8005	IF(JKONT(I).EQ.MODEIN) GO TO 8007	READ5990
	NOGO=1	READ6000
	WRITE(6,8008)II	READ6010
	GO TO 8007	READ6020
8006	IF(MODEIN.EQ.0.AND.JKONT(I).EQ.0) GO TO 8007	READ6030
	IF(JKONT(I).EQ.NOLEG) GO TO 8007	READ6040
	NOGO=1	READ6050
	WRITE(6,8008)II	READ6060
	GO TO 8007	READ6070
8012	IF(MODEIN.EQ.0 .AND.JKONT(I).EQ.0) GO TO 8007	READ6080
	IF(JKONT(I).EQ.2*NOLEG) GO TO 8007	READ6090
	NOGO=1	READ6100
	WRITE(6,8008)II	READ6110
	GO TO 8007	READ6120

8013	IF(MODEIN.EQ.0.AND.JKONT(I).EQ.0) GO TO 8007	READ6130
	IF(JKONT(I).EQ.NOOUT) GO TO 8007	READ6140
	NOGO=1	READ6150
	WRITE(6,8008)II	READ6160
8007	CONTINUE	READ6170
8008	FORMAT(/64H DATA DECK DOES NOT AGREE WITH CONTROL INFORMATION - C	READ6180
	*RD NUMBER,I5)	READ6190
	IF(NMODES.LE.MODEIN)GO TO 8011	READ6200
	NOGO=1	READ6210
	WRITE(6,8010)	READ6220
8010	FORMAT(/24H INCONSISTENT MODAL DATA)	READ6230
8011	CONTINUE	READ6240
	WRITE(6,8009)(NOCARD(I),I=1,MMM)	READ6250
8009	FORMAT(/27H NEW CARDS READ THIS RUN - 20(1X,I4)/(27X,20(1X,I4)))	READ6260
C	CALL DATAOT	READ6270
C	IF(NOGO.EQ.1) STOP	READ6280
	JTEST=1	READ6290
C	IF (IDSETN .EQ. 0) GO TO 4010	READ6300
C		READ6310
C	IF ANOTHER DATA CASE IS INDICATED (IDSETN .NE. 0) THEN SAVE	READ6320
C	BLANK COMMON AND THISV COMMON FOR REINITIALIZATION BEFORE	READ6330
C	THE NEXT DATA CASE IS READ	READ6340
C		READ6350
	REWIND 4	READ6360
	WRITE (4)(COMINT(I),I=1,1400),(TIMHSA(I),I=1,275)	READ6370
	ISAVCM = 1	READ6380
4010	CONTINUE	READ6390
C		READ6400
C	THIS PORTION OF THE PROGRAM INITIALIZES THE PROGRAM VARIABLES	READ6410
C		READ6420
C	DO NOT LET TIMAX .EQ. 0	READ6430
C		READ6440
	IF (TIMAX .EQ. 0.0) TIMAX = HMAX+HMAX	READ6450
	TIME=0.0	READ6460
	ISTAB=0	READ6470
	IBOTM=0	READ6480
	IF(INLEG.EQ.1)CUTERR=100.*GRAVE	READ6490
	IF(NFORC.NE.0) REWIND 3	READ6500
C		READ6510
C	INITIALIZATION FOR SUBROUTINE SOIL	READ6520
C		READ6530
	DO 62 I=1,3	READ6540
62	ATTHCK(I)=SS(1)-ATTHCK(I)	READ6550
	DO 63 I=1,NOLEG	READ6560
	IMPACT(I)=0	READ6570
	IPRTFP(I)=IPTCNT	READ6580
	KOUNT(I)=0	READ6590
63	ATTH(I)=SS(1)	READ6600
	ADIST=ATTHCK(3)	READ6610
	RADIAN=57.295779513	READ6620
	IF(NTYPE.NE.0) GO TO 68	READ6630
	SOILP(1)=SOILP(1)/RADIAN	READ6640
	TANPHI=TAN(SOILP(1))	READ6650
		READ6660
		READ6670
		READ6680

	CMS=29.*EXP(1.4*SOILP(3))*TANPHI	READ6690
	SLRHO=SOILP(2)/GRAVE	READ6700
	TANPH2=TAN(SOILP(1)/2.)	READ6710
	SOILNU=(3.14159265/3.)* (1.+TANPH2)/(1.-TANPH2)	READ6720
	IF(SOILP(3).GE.0.5) GO TO 67	READ6730
	CDCONT=(GRAV/GRAVE)*(4.+80.*SOILP(3))*TANPHI/(RAD(4)**2)	READ6740
	GO TO 68	READ6750
	67 CDCONT=4.*(GRAV/GRAVE)*EXP(4.83*SOILP(3))*TANPHI/(RAD(4)**2)	READ6760
	68 CONTINUE	READ6770
C		READ6780
C	INITIALIZE THE CENTER BODY MASS MATRIX	READ6790
C		READ6800
	DO 70 I = 1, 6	READ6810
	DO 70 J = 1, 6	READ6820
	70 AM(I,J) = 0.0	READ6830
	AM(1,1) = CBMASS	READ6840
	AM(2,2) = CBMASS	READ6850
	AM(3,3) = CBMASS	READ6860
	AM(4,4) = CBIXX	READ6870
	AM(5,5) = CBIYY	READ6880
	AM(6,6) = CBIZZ	READ6890
	AM(4,5) = -CBIXY	READ6900
	AM(4,6) = -CBIXZ	READ6910
	AM(5,6) = -CBIYZ	READ6920
	AM(5,4) = -CBIXY	READ6930
	AM(6,4) = -CBIXZ	READ6940
	AM(6,5) = -CBIYZ	READ6950
C		READ6960
C	INITIALIZATION FOR SUBROUTINE STRUT	READ6970
C		READ6980
C	STRUT LENGTHS	READ6990
C		READ7000
	DO 90 I=1,NOLEG	READ7010
	DX=XMSCB(I)-XFP(I)	READ7020
	DY=YMSCB(I)-YFP(I)	READ7030
	DZ=ZMSCB(I)-ZFP(I)	READ7040
	SLOMS(I)=SQRT(DX*DX+DY*DY+DZ*DZ)	READ7050
	SLNGMS(I)=SLOMS(I)	READ7060
	DO 90 J=1,2	READ7070
	NNN=2*(I-1)+J	READ7080
	IF(ILEG.EQ.0) GO TO 84	READ7090
	CDX=DX/SLOMS(I)	READ7100
	CDY=DY/SLOMS(I)	READ7110
	CDZ=DZ/SLOMS(I)	READ7120
	DEL=SLOMS(I)-DRAGST	READ7130
	DDX=XDSCB(NNN)-(XFP(I)+CDX*DEL)	READ7140
	DDY=YDSCB(NNN)-(YFP(I)+CDY*DEL)	READ7150
	DDZ=ZDSCB(NNN)-(ZFP(I)+CDZ*DEL)	READ7160
	GO TO 85	READ7170
	84 CONTINUE	READ7180
	DDX=XDSCB(NNN)-XFP(I)	READ7190
	DDY=YDSCB(NNN)-YFP(I)	READ7200
	DDZ=ZDSCB(NNN)-ZFP(I)	READ7210
	85 SLODS(NNN)=SQRT(DDX*DDX+DDY*DDY+DDZ*DDZ)	READ7220
	SLNGDS(NNN)=SLODS(NNN)	READ7230
	90 CONTINUE	READ7240

C		READ7250
C	MAIN STRUTS	READ7260
C		READ7270
	DO 87 I=1,NOLEG	READ7280
	STRPMS(I)=0.0	READ7290
	URCMS(I)=0.0	READ7300
	URTMS(I)=0.0	READ7310
	SETCMS(I)=0.0	READ7320
	SETTMS(I)=0.0	READ7330
	PRFCMS(I)=0.0	READ7340
	PRFTMS(I)=0.0	READ7350
	FRVMSC(I)=0.0	READ7360
	FRVMST(I)=0.0	READ7370
	IPOCMS(I)=1	READ7380
	IPOTMS(I)=1	READ7390
	INDCMS(I)=0	READ7400
	INDTMS(I)=0	READ7410
	IPRMS(I)=0	READ7420
	87 CONTINUE	READ7430
C		READ7440
C	DRAG STRUTS	READ7450
C		READ7460
	II=2*NOLEG	READ7470
	DO 86 I=1,II	READ7480
	STRPDS(I)=0.0	READ7490
	URCDS(I)=0.0	READ7500
	URTDS(I)=0.0	READ7510
	SETCDS(I)=0.0	READ7520
	SETTDS(I)=0.0	READ7530
	PRFCDS(I)=0.0	READ7540
	PRFTDS(I)=0.0	READ7550
	FRVDSC(I)=0.0	READ7560
	FRVDST(I)=0.0	READ7570
	IPOCDS(I)=1	READ7580
	IPOTDS(I)=1	READ7590
	INDCDS(I)=0	READ7600
	INDTDS(I)=0	READ7610
	IPRDS(I)=0	READ7620
	86 CONTINUE	READ7630
C		READ7640
C	CONVERT ANGULAR QUANTITIES TO RADIANS	READ7650
C		READ7660
	WX=WX/RADIAN	READ7670
	WY=WY/RADIAN	READ7680
	WZ=WZ/RADIAN	READ7690
	ZETA=ZETA/RADIAN	READ7700
	DO 88 I=1,NMODES	READ7710
	88 OMEGA(I)=OMEGA(I)*2.*3.14159265	READ7720
C		READ7730
C	INITIALIZE ANGULAR QUANTITIES	READ7740
C		READ7750
	COX=COS(ANGX/RADIAN)	READ7760
	SIX=SIN(ANGX/RADIAN)	READ7770
	COY=COS(ANGY/RADIAN)	READ7780
	SIY=SIN(ANGY/RADIAN)	READ7790
	COZ=COS(ANGZ/RADIAN)	READ7800

	SIZ=SIN(ANGZ/RADIAN)	READ7810
	SIZT=SIN(ZETA)	READ7820
	COZT=COS(ZETA)	READ7830
	THTA=ASIN(-SIZT*COZ*COY-COZT*(SIZ*SIX-COZ*SIY*COX))	READ7840
	AAN=-SIZT*SIZ*COY+COZT*(COZ*SIX+SIZ*SIY*COX)	READ7850
	BBD=SIZT*SIY+COZT*COY*COX	READ7860
	IF(ABS(BBD).LT.1.E-09)GO TO 104	READ7870
	PHI=ATAN2(AAN,BBD)	READ7880
	GO TO 101	READ7890
104	IF(AAN.GT.0.0)PHI=+3.14159265/2.	READ7900
	IF(AAN.LT.0.0)PHI=-3.14159265/2.	READ7910
101	CONTINUE	READ7920
	AAN=SIZ*COX+COZ*SIY*SIX	READ7930
	BBD=COZT*COZ*COY-SIZT*(SIZ*SIX-COZ*SIY*COX)	READ7940
	IF(ABS(BBD).LT.1.E-09)GO TO 102	READ7950
	PSI=ATAN2(AAN,BBD)	READ7960
	GO TO 103	READ7970
102	IF(AAN.GT.0.0)PSI=+3.14159265/2.	READ7980
	IF(AAN.LT.0.0)PSI=-3.14159265/2.	READ7990
103	CONTINUE	READ8000
C		READ8010
C	INITIALIZE VELOCITIES	READ8020
C		READ8030
	ZETA=-ZETA	READ8035
	GSINZT=SIN(ZETA)*GRAV	READ8040
	GCOSZT=COS(ZETA)*GRAV	READ8050
	XSD=VELX*COS(ZETA)+VELZ*SIN(ZETA)	READ8060
	YSD=VELY	READ8070
	ZSD=-VELX*SIN(ZETA)+VELZ*COS(ZETA)	READ8080
	RETURN	READ8090
	END	READ8100

	SUBROUTINE DATAOT	DATA 10
C		DATA 20
C	THIS SUBROUTINE LISTS THE INPUT DATA	DATA 30
C		DATA 40
	DIMENSION NO(100),NUMB(25)	DATA 50
C		DATA 60
	COMMON / THISV/ TIMHSA(1), ATTH(5),AM(6,6),AME1(3),INDFPI(6),	DATA 70
	1 INDFPC(6),	DATA 80
	2 STRPDS(10), STRPMS(5), IPOCD(10), IPOCMS(5), URCDS (10),	DATA 90
	3 URTDS (10), URCMS (5), URTMS (5), SETCDS(10), SETTDS(10),	DATA 100
	4 SETCMS(5), SETTMS(5), INDCDS(10), INDTDS(10), INDCMS(10),	DATA 110
	5 INDTMS(5), PRFCDS (10), PRFTDS (10), PRFCMS (5), PRFTMS (5),	DATA 120
	6 IPRDS (10), IPRMS (5), FRVDSC(10), FRVDST(10), FRVMSC(5),	DATA 130
	L FRVMST(5), IPOTDS(10), IPOTMS(5)	DATA 140
	EQUIVALENCE (NFTPDS, NOLEG)	DATA 150
	COMMON COMINT(400)	DATA 160
	EQUIVALENCE (COMINT(4), XSD)	DATA 170
	EQUIVALENCE (COMINT(16), YSD)	DATA 180
	EQUIVALENCE (COMINT(28), ZSD)	DATA 190
	EQUIVALENCE (COMINT(36), PHI)	DATA 200
	EQUIVALENCE (COMINT(44), WX)	DATA 210
	EQUIVALENCE (COMINT(52), THTA)	DATA 220
	EQUIVALENCE (COMINT(60), WY)	DATA 230
	EQUIVALENCE (COMINT(68), PSI)	DATA 240
	EQUIVALENCE (COMINT(76), WZ)	DATA 250
	EQUIVALENCE (COMINT(324), TIME)	DATA 260
	EQUIVALENCE (COMINT(325), HMAX)	DATA 270
	EQUIVALENCE (COMINT(326), HMIN)	DATA 280
	EQUIVALENCE (COMINT(327), EMIN)	DATA 290
	EQUIVALENCE (COMINT(328), EMAX)	DATA 300
	EQUIVALENCE (COMINT (337), HZ)	DATA 310
	EQUIVALENCE (COMINT(338), CUTERR)	DATA 320
	EQUIVALENCE (COMINT(339), IP)	DATA 330
	EQUIVALENCE (COMINT(340), IVARH)	DATA 340
	EQUIVALENCE (COMINT(341), IMTH)	DATA 350
	EQUIVALENCE (COMINT(342), IPRNT)	DATA 360
	EQUIVALENCE (COMINT(343), IFIN)	DATA 370
	EQUIVALENCE (COMINT(361), IPTCNT)	DATA 380
	EQUIVALENCE (COMINT (362), IDSETN)	DATA 390
	EQUIVALENCE (COMINT (1399), NOCASE)	DATA 400
	COMMON	DATA 410
	1 CBMASS, CBIXX , CBIXZ , CBIYY , CBIYZ , CBIZZ , FPMASS,CBIXY,	DATA 420
	2 DC(3,3) , XFP(5), YFP(5), ZFP(5), WNX(5), WNY(5),	DATA 430
	3 WNZ(5), PX(5) , PY(5) , PZ(5) , GM(5) , OMEGA(5) ,	DATA 440
	4 GRAV , GRAVE , ZETA , FTS (6) , FSXSI(5) ,	DATA 450
	5 FSYSI(5) , FSZSI(5) , SOILX(5) ,	DATA 460
	6 SOILY(5) , SOILZ(5) , PMSX(5,5) ,	DATA 470
	7 PMSY(5,5) , PMSZ(5,5) , PDSX(10,5) ,	DATA 480
	8 PDSY(10,5) , PDSZ(10,5) , FLXS , FLYS , FLZS ,	DATA 490
	9 TLXL , TLYL , TLZL , SLO , XMSCB(5) ,	DATA 500
C	YMSCB(5) , ZMSCB(5) , XDSCB(10) ,	DATA 510
D	YDSCB(10) , ZDSCB(10) , ILEG , IMS ,	DATA 520
E	FSTX , FSTY , FSTZ , PVCBX , PVCBY , PVCBZ ,	DATA 530
F	PVFPX , PVFPY , PVFPZ , NOLEG , SLOMS(5) ,	DATA 540
G	SLODS(10)	DATA 550

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COMMON DATA 560
1 PFCMS(5) , PFCDS(5) , PFTDS(5) , DATA 570
2 PFTMS(5) , SRCDS(5) , SRCMS(5) , DATA 580
3 SRTDS(5) , SRTMS(5) , COEFDS, COEFMS, GAMDS , DATA 590
4 GAMMS , SRUCDS, SRUCMS, SRUTDS, SRUTMS, SCMXDS, SCMXMS, DATA 600
5 STMXDS, STMXMS, CDCDS(5) , CDCMS(5) , DATA 610
6 CDTDS(5) , CDTMS(5) , FRICDS, FRICMS, IRETDS, DATA 620
7 IRETMS, STROKE(10) , STRKDS(10) , STRKMS( 5) , DATA 630
8 LENGTH,CDXS,CDYS,CDZS , NTYPE , RAD(4), SS(4), ATTHCK(3), DATA 640
9 ATTPRS(3) , ADIST , SOILP(3). , NMODES, DATA 650
A COEF , GAMMA , FRIC , SCMAX , STMAX , CDC(5), CDT(5), DATA 660
B SRULT , SRULC , SRT(5), SRC(5), PFT(5), PFC(5), GFLEGS(5), DATA 670
C CURMSV, CURMSL, INDFXD, INDFYD, INDFZD, INDFXR, INDFYR, DATA 680
D INDFZR, TIMAX , DRAGST, IFP DATA 690
COMMON DATA 700
1CMS,CDCONT,SOILNU,SLRHO, NOOUT,XOUT(10),YOUT(10),ZOUT(10), DATA 710
2MODEIN,POUTX(10,5),POUTY(10,5),POUTZ(10,5),PCGX(5),PCGY(5), DATA 720
3PCGZ(5),AEI1,AEI2,FORMS(5),FORDS(10),FORCE,NFORC,SAVMSX(5), DATA 730
4SAVMSZ(5),SAVDSX(10),SAVDSY(10),SAVDSZ(10),IQUOUT,GSINZT, DATA 740
5GCOSZT,STAB,STABVL,ISTAB,JCKSAB,VELX,VELY,VELZ,SAVMSY(5) DATA 750
COMMON DATA 760
1 SMXMSC(5), TMXMSC(5), SMXMST(5), TMXMST(5), DATA 770
2 SMXDSC(10), TMXDSC(10), SMXDST(10), TMXDST(10) DATA 780
3 ,SLNGMS(5) ,SLNGDS(10), CURDSL, INLEG , IFPRT, DATA 790
4 IMPACT(5) ,IPRTFP(5), KOUNT(5) ,ANGX, ANGY, ANGZ DATA 800
C DATA 810
DO 5000 I=1,100 DATA 820
5000 NO(I)=I DATA 830
DO 5001 I=1,25 DATA 840
5001 NUMB(I)=I*100 DATA 850
C DATA 860
C PROGRAM CONTROL DATA DATA 870
C DATA 880
WRITE(6,101)NO(1),NOCASE DATA 890
WRITE(6,102)NO(2),TIMAX,IPTCNT,INLEG,IFPRT DATA 900
WRITE(6,103)NO(3),NMODES,NOOUT DATA 910
WRITE(6,104)NO(4),INDFXD,INDFYD,INDFZD DATA 920
WRITE(6,105)NO(5),INDFXR,INDFYR,INDFZR DATA 930
WRITE(6,106)NO(6),HMAX,HMIN,EMAX,EMIN,IP DATA 940
WRITE(6,107)NO(7),IVARH,IMTH,CUTERR DATA 950
WRITE(6,108)NO(8),NFORC,IQUOUT,JCKSAB,IDSETN DATA 960
IF(NFORC.NE.0.AND.IDSETN.NE.0)WRITE(6,3000) DATA 970
C DATA 980
C INITIAL CONDITIONS DATA 990
C DATA1000
WRITE(6,109)NO(9),ZETA,GRAV,GRAVE DATA1010
WRITE(6,110)NO(10),ANGX,ANGY,ANGZ DATA1020
WRITE(6,111)NO(11),WX,WY,WZ DATA1030
WRITE(6,112)NO(12),VELX,VELY,VELZ DATA1040
C DATA1050
C CENTER BODY DATA DATA1060
C DATA1070
WRITE(6,113)NO(13),CBMASS DATA1080
WRITE(6,114)NO(14),CBIXX,CBIYY,CBIZZ DATA1090
WRITE(6,115)NO(15),CBIXY,CBIXZ,CBIYZ DATA1100
IF(NOOUT.EQ.0)GO TO 2 DATA1110

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	DO 1 I=1,NOOUT	DATA1120
	1 WRITE(6,1100)NUMB(1),I,XOUT(I),YOUT(I),ZOUT(I)	DATA1130
	GO TO 3	DATA1140
	2 WRITE(6,1101)	DATA1150
	3 CONTINUE	DATA1160
C		DATA1170
C	FOOTPAD DATA	DATA1180
C		DATA1190
	WRITE(6,116)NO(16),FPMASS	DATA1200
	WRITE(6,117)NO(17),(RAD(I),I=1,4)	DATA1210
	WRITE(6,118)NO(18),(SS(I),I=1,4)	DATA1220
	WRITE(6,119)NO(19),(ATTHCK(I),I=1,3)	DATA1230
	WRITE(6,120)NO(20),(ATTPRS(I),I=1,3)	DATA1240
C		DATA1250
C	SOIL DATA	DATA1260
C		DATA1270
	IF(NTYPE.EQ.0)WRITE(6,121)NO(21),NTYPE	DATA1280
	IF(NTYPE.NE.0)WRITE(6,221)NO(21),NTYPE	DATA1290
	WRITE(6,122)NO(22),(SOILP(I),I=1,3)	DATA1300
C		DATA1310
C	LEG DATA	DATA1320
C		DATA1330
	WRITE(6,123)NO(23),NOLEG,ILEG,DRAGST	DATA1340
	DO 4 I=1,NOLEG	DATA1350
	4 WRITE(6,1200)NUMB(2),I,XFP(I),YFP(I),ZFP(I)	DATA1360
	DO 5 I=1,NOLEG	DATA1370
	5 WRITE(6,1300)NUMB(3),I,XMSCB(I),YMSCB(I),ZMSCB(I)	DATA1380
	WRITE(6,124)NO(24),(PFCMS(I),I=1,5)	DATA1390
	WRITE(6,125)NO(25),(PFTMS(I),I=1,5)	DATA1400
	WRITE(6,126)NO(26),(CDCMS(I),I=1,5)	DATA1410
	WRITE(6,127)NO(27),(CDTMS(I),I=1,5)	DATA1420
	WRITE(6,128)NO(28),(SRCMS(I),I=1,5)	DATA1430
	WRITE(6,129)NO(29),(SRTMS(I),I=1,5)	DATA1440
	WRITE(6,130)NO(30),SCMXMS,STMXMS,SRUCMS,SRUTMS	DATA1450
	WRITE(6,131)NO(31),IRETMS	DATA1460
	WRITE(6,132)NO(32),FRICMS,COEFMS,GAMMS,AEI1,AEI2	DATA1470
	II=2*NOLEG	DATA1480
	DO 6 I=1,II	DATA1490
	6 WRITE(6,1400)NUMB(4),I,XDSCB(I),YDSCB(I),ZDSCB(I)	DATA1500
	WRITE(6,133)NO(33),(PFCDS(I),I=1,5)	DATA1510
	WRITE(6,134)NO(34),(PFTDS(I),I=1,5)	DATA1520
	WRITE(6,135)NO(35),(CDCDS(I),I=1,5)	DATA1530
	WRITE(6,136)NO(36),(CDTDS(I),I=1,5)	DATA1540
	WRITE(6,137)NO(37),(SRCDS(I),I=1,5)	DATA1550
	WRITE(6,138)NO(38),(SRTDS(I),I=1,5)	DATA1560
	WRITE(6,139)NO(39),SCMXDS,STMXDS,SRUCDS,SRUTDS	DATA1570
	WRITE(6,140)NO(40),IRETDS	DATA1580
	WRITE(6,141)NO(41),FRICDS,COEFDS,GAMDS	DATA1590
C		DATA1600
C	MODAL DATA	DATA1610
C		DATA1620
	WRITE(6,142)NO(42),MODEIN	DATA1630
	IF(NMODES.EQ.0)WRITE(6,242)	DATA1640
	IF(MODEIN.EQ.0)RETURN	DATA1650
	DO 8 I=1,MODEIN	DATA1660
	8 WRITE(6,1500)NUMB(5),I,GM(I),OMEGA(I)	DATA1670

DO 9 I=1,MODEIN	DATA1680
9 WRITE(6,1600)NUMB(6),I,WNX(I),WNY(I),WNZ(I)	DATA1690
DO 10 I=1,MODEIN	DATA1700
10 WRITE(6,1700)NUMB(7),I,PX(I),PY(I),PZ(I)	DATA1710
DO 11 I=1,NOLEG	DATA1720
11 WRITE(6,1800)NUMB(8),I,(PMSX(I,J),J=1,MODEIN)	DATA1730
DO 12 I=1,NOLEG	DATA1740
12 WRITE(6,1900)NUMB(9),I,(PMSY(I,J),J=1,MODEIN)	DATA1750
DO 13 I=1,NOLEG	DATA1760
13 WRITE(6,2000)NUMB(10),I,(PMSZ(I,J),J=1,MODEIN)	DATA1770
DO 14 I=1,II	DATA1780
14 WRITE(6,2100)NUMB(11),I,(PDSX(I,J),J=1,MODEIN)	DATA1790
DO 15 I=1,II	DATA1800
15 WRITE(6,2200)NUMB(12),I,(PDSY(I,J),J=1,MODEIN)	DATA1810
DO 16 I=1,II	DATA1820
16 WRITE(6,2300)NUMB(13),I,(PDSZ(I,J),J=1,MODEIN)	DATA1830
IF(NOOUT.EQ.0)GO TO 21	DATA1840
DO 17 I=1,NOOUT	DATA1850
17 WRITE(6,2400)NUMB(14),I,(POUTX(I,J),J=1,MODEIN)	DATA1860
DO 18 I=1,NOOUT	DATA1870
18 WRITE(6,2500)NUMB(15),I,(POUTY(I,J),J=1,MODEIN)	DATA1880
DO 19 I=1,NOOUT	DATA1890
19 WRITE(6,2600)NUMB(16),I,(POUTZ(I,J),J=1,MODEIN)	DATA1900
21 CONTINUE	DATA1910
DO 20 I=1,MODEIN	DATA1920
20 WRITE(6,2700)NUMB(17),I,PCGX(I),PCGY(I),PCGZ(I)	DATA1930
RETURN	DATA1940
	DATA1950
C	DATA1960
C	DATA1970
C	DATA1980
101 FORMAT(5H CARD/2X,2HNO//10X,28H* * PROGRAM CONTROL DATA * *//,	DATA1990
* I4,10H CASE NO =,I6)	DATA2000
102 FORMAT(I4,9H TIMAX =,E10.3,10H IPTCNT =,I2,9H INLEG =,I2,	DATA2010
* 10H IFPPRT =,I2)	DATA2020
103 FORMAT(I4,10H NMODES =,I2, 9H NOOUT =,I2)	DATA2030
104 FORMAT(I4,10H INDFXD =,I2,10H INDFYD =,I2,10H INDFZD =,I2)	DATA2040
105 FORMAT(I4,10H INDFXR =,I2,10H INDFYR =,I2,10H INDFZR =,I2)	DATA2050
106 FORMAT(I4, 8H HMAX =,E10.3, 8H HMIN =,E10.3, 8H EMAX =,E10.3,	DATA2060
* 8H EMIN =,E10.3,6H IP =,I2)	DATA2070
107 FORMAT(I4,9H IVARH =,I2,8H IMTH =,I2,10H CUTERR =,E10.3)	DATA2080
108 FORMAT(I4,9H NFORC =,I2,10H IQUOUT =,I2,10H JCKSAB =,I2,	DATA2090
*10H IDSETN =,I2)	DATA2100
109 FORMAT(/10X,26H* * INITIAL CONDITIONS * *//,I4, 8H ZETA =,E10.3	DATA2110
* , 8H GRAV =,E10.3, 9H GRAVE =,E10.3)	DATA2120
110 FORMAT(I4,8H ANGX =,E10.3,8H ANGY =,E10.3,8H ANGZ =,E10.3)	DATA2130
111 FORMAT(I4,6H WX =,E10.3,6H WY =,E10.3,6H WZ =,E10.3)	DATA2140
112 FORMAT(I4,8H VELX =,E10.3,8H VELY =,E10.3,8H VELZ =,E10.3)	DATA2150
113 FORMAT(/10X,24H* * CENTER BODY DATA * *//,I4,10H CBMASS =,	DATA2160
* E10.3)	DATA2170
114 FORMAT(I4,9H CBIXX =,E10.3,9H CBIYY =,E10.3,9H CBIZZ =,E10.3)	DATA2180
115 FORMAT(I4,9H CBIXY =,E10.3,9H CBIXZ =,E10.3,9H CBIYZ =,E10.3)	DATA2190
1100 FORMAT(I4,5H I =,I2,8H XOUT =,E10.3,8H YOUT =,E10.3,8H ZOUT =,	DATA2200
* E10.3)	DATA2210
1101 FORMAT(/5X,26HNO SECONDARY OUTPUT POINTS)	DATA2220
116 FORMAT(/10X,20H* * FOOTPAD DATA * *//I4,10H FPMASS =,E10.3)	DATA2230
117 FORMAT(I4,10H RAD(I) =,4(2X,E10.3))	

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118 FORMAT(I4,9H SS(I) = ,4(2X,E10.3)) DATA2240
119 FORMAT(I4,13H ATTHCK(I) = ,3(2X,E10.3)) DATA2250
120 FORMAT(I4,13H ATTPRS(I) = ,3(2X,E10.3)) DATA2260
121 FORMAT(/10X,17H* * SOIL DATA * *,//,I4, 9H NTYPE = ,I2,/,4X, DATA2270
* 25H (PRIMARY SOIL MECHANICS)) DATA2280
221 FORMAT(/10X,17H* * SOIL DATA * *,//,I4,9H NTYPE = ,I2,/,4X, DATA2290
* 27H (SECONDARY SOIL MECHANICS)) DATA2300
122 FORMAT(I4,12H SOILP(I) = ,E10.3,3H , ,E10.3,3H , ,E10.3) DATA2310
123 FORMAT(/10X,16H* * LEG DATA * *,//,I4, 9H NOLEG = ,I2, DATA2320
* 8H ILEG = ,I2,10H DRAGST = ,E10.3) DATA2330
1200 FORMAT(I4,5H I = ,I2,7H XFP = ,E10.3,7H YFP = ,E10.3,7H ZFP = , DATA2340
* E10.3) DATA2350
1300 FORMAT(I4,5H I = ,I2,9H XMSCB = ,E10.3,9H YMSCB = ,E10.3, DATA2360
* 9H ZMSCB = ,E10.3) DATA2370
124 FORMAT(I4,9H PFCMS = ,5(2X,E10.3)) DATA2380
125 FORMAT(I4,9H PFTMS = ,5(2X,E10.3)) DATA2390
126 FORMAT(I4,9H CDCMS = ,5(2X,E10.3)) DATA2400
127 FORMAT(I4,9H CDTMS = ,5(2X,E10.3)) DATA2410
128 FORMAT(I4,9H SRCMS = ,5(2X,E10.3)) DATA2420
129 FORMAT(I4,9H SRTMS = ,5(2X,E10.3)) DATA2430
130 FORMAT(I4,10H SCMAMS = ,E10.3,10H STMAMS = ,E10.3,10H SRUCMS = , DATA2440
* E10.3,10H SRUTMS = ,E10.3) DATA2450
131 FORMAT(I4,10H IRETMS = ,I2) DATA2460
132 FORMAT(I4,10H FRICMS = ,E10.3,10H COEFMS = ,E10.3,9H GAMMS = , DATA2470
* E10.3,8H AEI1 = ,E10.3,8H AEI2 = ,E10.3) DATA2480
1400 FORMAT(I4,5H I = ,I2,9H XDSCB = ,E10.3,9H YDSCB = ,E10.3, DATA2490
* 9H ZDSCB = ,E10.3) DATA2500
133 FORMAT(I4,9H PFCDS = ,5(2X,E10.3)) DATA2510
134 FORMAT(I4,9H PFTDS = ,5(2X,E10.3)) DATA2520
135 FORMAT(I4,9H CDCDS = ,5(2X,E10.3)) DATA2530
136 FORMAT(I4,9H CDTDS = ,5(2X,E10.3)) DATA2540
137 FORMAT(I4,9H SRCDS = ,5(2X,E10.3)) DATA2550
138 FORMAT(I4,9H SRTDS = ,5(2X,E10.3)) DATA2560
139 FORMAT(I4,10H SCMXDS = ,E10.3,10H STMXDS = ,E10.3,10H SRUCDS = , DATA2570
* E10.3,10H SRUTDS = ,E10.3) DATA2580
140 FORMAT(I4,10H IRETDS = ,I2) DATA2590
141 FORMAT(I4,10H FRICDS = ,E10.3,10H COEFDS = ,E10.3,9H GAMDS = , DATA2600
* E10.3) DATA2610
142 FORMAT(/10X,18H* * MODAL DATA * *,//,I5,10H MODEIN = ,I2) DATA2620
242 FORMAT(/5X,25HRIGID CENTER BODY ASSUMED) DATA2630
1500 FORMAT(I5,5H I = ,I2,6H GM = ,E10.3,9H OMEGA = ,E10.3) DATA2640
1600 FORMAT(I5,5H I = ,I2,7H WNX = ,E10.3,7H WNY = ,E10.3,7H WNZ = , DATA2650
* E10.3) DATA2660
1700 FORMAT(I5,5H I = ,I2,6H PX = ,E10.3,6H PY = ,E10.3,6H PZ = ,E10.3) DATA2670
1800 FORMAT(I5,5H I = ,I2,8H PMSX = ,5(2X,E10.3)) DATA2680
1900 FORMAT(I5,5H I = ,I2,8H PMSY = ,5(2X,E10.3)) DATA2690
2000 FORMAT(I5,5H I = ,I2,8H PMSZ = ,5(2X,E10.3)) DATA2700
2100 FORMAT(I5,5H I = ,I2,8H PDSX = ,5(2X,E10.3)) DATA2710
2200 FORMAT(I5,5H I = ,I2,8H PDSY = ,5(2X,E10.3)) DATA2720
2300 FORMAT(I5,5H I = ,I2,8H PDSZ = ,5(2X,E10.3)) DATA2730
2400 FORMAT(I5,5H I = ,I2,9H POUTX = ,5(2X,E10.3)) DATA2740
2500 FORMAT(I5,5H I = ,I2,9H POUTY = ,5(2X,E10.3)) DATA2750
2600 FORMAT(I5,5H I = ,I2,9H POUTZ = ,5(2X,E10.3)) DATA2760
2700 FORMAT(I5,5H I = ,I2,8H PCGX = ,E10.3,8H PCGY = ,E10.3,8H PCGZ = , DATA2770
* E10.3) DATA2780
3000 FORMAT(/5X,50HWARNING - PROGRAM CONTROL DATA INDICATES THIS CASE,/ DATA2790

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* 55HIS PART OF A MULTIPLE DATA SET RUN AND A SECONDARY TIME, /DATA2800
* 52HHISTORY TAPE IS TO BE GENERATED. THIS COMBINATION OF, / DATA2810
* 42HPROGRAM OPTIONS MUST BE HANDLED WITH CARE.,) DATA2820
END DATA2830

OVERLAY (LLMPT5, 2, 0)

OVER 30

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PROGRAM LLMPEX
COMMON / IZZZRM / IPTATL, IPTOTL, LOCNAM(90)
DIMENSION XSI(8), IAD(8), IND(8)
EQUIVALENCE ( TIMHSA(1), LTIMHS )
DIMENSION TIMHSC(275)
DIMENSION TIMHSB(275)
DIMENSION IMHSA(1)
EQUIVALENCE ( IMHSA(1), TIMHSA(1) )
DIMENSION GC(4,5), GCD(4,5), GCDD(4,5)
DIMENSION XFPS(4,5), YFPS(4,5), ZFPS(4,5),
1 XFPSD(4,5), YFPSD(4,5), ZFPSD(4,5),
2 XFPSDD(4,5), YFPSDD(4,5), ZFPSDD(4,5)
DIMENSION XMN1(8)
DIMENSION INOUTV(5)
COMMON / THISV/ TIMHSA(1), ATTH(5), AM(6,6), AME1(3), INDFPI(6),
1 INDFPC(6),
2 STRPDS(10), STRPMS( 5), IPOCD(10), IPOCMS( 5), URCDS (10),
3 URTDS (10), URCMS ( 5), URTMS ( 5), SETCDS(10), SETTDS(10),
4 SETCMS( 5), SETTMS( 5), INDCDS(10), INDTDS(10), INDCMS(10),
5 INDTMS( 5), PRFCDS (10), PRFTDS (10), PRFCMS ( 5), PRFTMS ( 5),
6 IPRDS (10), IPRMS ( 5), FRVDSC(10), FRVDST(10), FRVMSC( 5),
L FRVMST( 5), IPOTDS(10), IPOTMS( 5)
EQUIVALENCE ( NFTPDS, NOLEG )
COMMON COMINT(400)
EQUIVALENCE ( COMINT( 3 ), IBOTM )
EQUIVALENCE ( COMINT( 4 ), XSD )
EQUIVALENCE ( COMINT( 8 ), XSDD )
EQUIVALENCE ( COMINT( 12 ), YS )
EQUIVALENCE ( COMINT( 16 ), YSD )
EQUIVALENCE ( COMINT( 20 ), YSDD )
EQUIVALENCE ( COMINT( 24 ), ZS )
EQUIVALENCE ( COMINT( 28 ), ZSD )
EQUIVALENCE ( COMINT( 32 ), ZSDD )
EQUIVALENCE ( COMINT( 36 ), PHI )
EQUIVALENCE ( COMINT( 40 ), PHID )
EQUIVALENCE ( COMINT( 44 ), WX )
EQUIVALENCE ( COMINT( 48 ), WXD )
EQUIVALENCE ( COMINT( 52 ), THTA )
EQUIVALENCE ( COMINT( 56 ), THTAD )
EQUIVALENCE ( COMINT( 60 ), WY )
EQUIVALENCE ( COMINT( 64 ), WYD )
EQUIVALENCE ( COMINT( 68 ), PSI )
EQUIVALENCE ( COMINT( 72 ), PSID )
EQUIVALENCE ( COMINT( 76 ), WZ )
EQUIVALENCE ( COMINT( 80 ), WZD )
EQUIVALENCE ( COMINT( 84 ), GC )
EQUIVALENCE ( COMINT(104), GCD )
EQUIVALENCE ( COMINT(124), GCDD )
EQUIVALENCE ( COMINT(144), XFPS )
EQUIVALENCE ( COMINT(164), XFPSD )
EQUIVALENCE ( COMINT(184), XFPSDD )
EQUIVALENCE ( COMINT(204), YFPS )
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LLEX 10
LLEX 20
LLEX 30
LLEX 40
LLEX 50
LLEX 60
LLEX 70
LLEX 80
LLEX 90
LLEX 100
LLEX 110
LLEX 120
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LLEX 140
LLEX 150
LLEX 160
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LLEX 450
LLEX 460
LLEX 470
LLEX 480
LLEX 490
LLEX 500
LLEX 510
LLEX 520
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EQUIVALENCE (COMINT(224), YFPSD)	LLEX 530
EQUIVALENCE (COMINT(244), YFPSDD)	LLEX 540
EQUIVALENCE (COMINT(264), ZFPS)	LLEX 550
EQUIVALENCE (COMINT(284), ZFPSD)	LLEX 560
EQUIVALENCE (COMINT(304), ZFPSDD)	LLEX 570
EQUIVALENCE (COMINT(324), TIME)	LLEX 580
EQUIVALENCE (COMINT(325), HMAX)	LLEX 590
EQUIVALENCE (COMINT(326), HMIN)	LLEX 600
EQUIVALENCE (COMINT(327), EMIN)	LLEX 610
EQUIVALENCE (COMINT(328), EMAX)	LLEX 620
EQUIVALENCE (COMINT(329), XSI)	LLEX 630
EQUIVALENCE (COMINT (337), HZ)	LLEX 640
EQUIVALENCE (COMINT(338), CUTERR)	LLEX 650
EQUIVALENCE (COMINT(339), IP)	LLEX 660
EQUIVALENCE (COMINT(340), IVARH)	LLEX 670
EQUIVALENCE (COMINT(341), IMTH)	LLEX 680
EQUIVALENCE (COMINT(342), IPRNT)	LLEX 690
EQUIVALENCE (COMINT(343), IFIN)	LLEX 700
EQUIVALENCE (COMINT(344), IAD)	LLEX 710
EQUIVALENCE (COMINT(352), IND)	LLEX 720
EQUIVALENCE (COMINT(360), JCUT)	LLEX 730
EQUIVALENCE (COMINT(361), IPTCNT)	LLEX 740
EQUIVALENCE (COMINT(363), IVAL)	LLEX 750
EQUIVALENCE (COMINT(364), XS)	LLEX 760
C COMINT(364-367) USED BY XS	LLEX 770
EQUIVALENCE (COMINT(368), CURDT)	LLEX 780
EQUIVALENCE (COMINT(369), XMNI)	LLEX 790
C XMNI USES COMINT(369-376)	LLEX 800
COMMON	LLEX 810
1 CBMASS, CBIXX , CBIXZ , CBIYY , CBIYZ , CBIZZ , FPMASS,CBIXY,	LLEX 820
2 DC(3,3) , XFP(5), YFP(5), ZFP(5), WNX(5), WNY(5),	LLEX 830
3 WNZ(5), PX(5) , PY(5) , PZ(5) , GM(5) , OMEGA(5) ,	LLEX 840
4 GRAV , GRAVE , ZETA , FTS (6) , FSXSI(5) ,	LLEX 850
5 FSYSI(5) , FSZSI(5) , SOILX(5) ,	LLEX 860
6 SOILY(5) , SOILZ(5) , PMSX(5,5) ,	LLEX 870
7 PMSY(5,5) , PMSZ(5,5) , PDSX(10,5) ,	LLEX 880
8 PDSY(10,5) , PDSZ(10,5) , FLXS , FLYS , FLZS ,	LLEX 890
9 TLXL , TLYL , TLZL , SLO , XMSCB(5) ,	LLEX 900
C YMSCB(5) , ZMSCB(5) , XDSCB(10) ,	LLEX 910
D YDSCB(10) , ZDSCB(10) , ILEG , IMS ,	LLEX 920
E FSTX , FSTY , FSTZ , PVCBX , PVCBY , PVCBZ ,	LLEX 930
F PVFPX , PVFPY , PVFPZ , NOLEG , SLOMS(5) ,	LLEX 940
G SLODS(10)	LLEX 950
COMMON	LLEX 960
1 PFCMS(5) , PFCDS(5) , PFTDS(5) ,	LLEX 970
2 PFTMS(5) , SRCDS(5) , SRCMS(5) ,	LLEX 980
3 SRTDS(5) , SRTMS(5) , COEFDS, COEFMS, GAMDS ,	LLEX 990
4 GAMMS , SRUCDS, SRUCMS, SRUTDS, SRUTMS, SCMxDs, SCMxMS,	LLEX1000
5 STMxDs, STMxMS, CDCDS(5) , CDCMS(5) ,	LLEX1010
6 CDTDS(5) , CDTMS(5) , FRICDS, FRICMS, IRETDS,	LLEX1020
7 IRETMS, STROKE(10) , STRKDS(10) , STRKMS(5) ,	LLEX1030
8 LENGTH,CDXS,CDYS,CDZS , NTYPE , RAD(4), SS(4) , ATTHCK(3),	LLEX1040
9 ATTPRS(3) , ADIST , SOILP(3) , NMODES,	LLEX1050
A COEF , GAMMA , FRIC , SCMAX , STMAX , CDC(5), CDT(5),	LLEX1060
B SRULT , SRULC , SRT(5), SRC(5), PFT(5), PFC(5), GFLEGS(5),	LLEX1070
C CURMSV, CURMSL, INDFXD, INDFYD, INDFZD, INDFXR, INDFYR,	LLEX1080

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D      INDFZR, TIMAX , DRAGST, IFP                                LLEX1090
COMMON                                                            LLEX1100
1CMS, CDCONT, SOILNU, SLRHO,      NOOUT, XOUT(10), YOUT(10), ZOUT(10), LLEX1110
2MODEIN, POUTX(10,5), POUTY(10,5), POUTZ(10,5), PCGX(5), PCGY(5), LLEX1120
3PCGZ(5), AEI1, AEI2, FORMS(5), FORDS(10), FORCE, NFORC, SAVMSX(5), LLEX1130
4SAVMSZ(5), SAVDSX(10), SAVDSY(10), SAVDSZ(10), IQUOUT, GSINZT, LLEX1140
5GCOSZT, STAB, STABVL, ISTAB, JCKSAB, VELX, VELY, VELZ, SAVMSY(5) LLEX1150
COMMON                                                            LLEX1160
1      SMXMSC(5), TMXMSC(5), SMXMST(5), TMXMST(5), LLEX1170
2      SMXDSC(10), TMXDSC(10), SMXDST(10), TMXDST(10) LLEX1180
3      ,SLNGMS(5), SLNGDS(10), CURDSL, INLEG , IFPRT, LLEX1190
4      IMPACT(5), IPRTFP(5), KOUNT(5) ,ANGX, ANGY, ANGZ LLEX1200
DIMENSION AHOLD(6), AMINV(6,6), FLTL(6) LLEX1210
DIMENSION GF(5), GFO(5) LLEX1220
DIMENSION AMFP(5), FO(3), FE(6) LLEX1230
DIMENSION AME2(3) LLEX1240
C      PRECALCULATE SOME CONSTANTS USED LATER IN MODE WORK LLEX1250
DIMENSION PYZ(5), PYZ2(5), PXZ(5), PXZ2(5), PXY(5), PXY2(5), LLEX1260
1      WNZNY(5), WNXNZ(5), WNYNX(5), PYNZ2(5), PZNX2(5), PXNY2(5), W2GM(5) LLEX1270
C      SET INTEGRATION LIST UPDATE INDICATOR LLEX1280
INDIVR = 0 LLEX1290
LTIMHS = 273 LLEX1300
C      PRINT COUNT LLEX1310
KPTCNT = 99999 LLEX1320
C      PITCH ANGLE LIMIT LLEX1330
ATIPMX = 1.55 LLEX1340
C      TIME HISTORY UPDATE INDICATOR LLEX1350
INDBPU = 1 LLEX1360
C      INTEGRATION LOOP FIRST PASS INDICATOR LLEX1370
INDIFP = 0 LLEX1380
IF ( NMODES .EQ. 0 ) GO TO 110 LLEX1390
DO 100 I = 1, NMODES LLEX1400
PYZ(I) = PY(I)+PZ(I) LLEX1410
PYZ2(I) = PYZ(I)+PYZ(I) LLEX1420
PXZ(I) = PX(I)+PZ(I) LLEX1430
PXZ2(I) = PXZ(I)+PXZ(I) LLEX1440
PXY(I) = PX(I)+PY(I) LLEX1450
PXY2(I) = PXY(I)+PXY(I) LLEX1460
WNZNY(I)= WNZ(I)-WNY(I) LLEX1470
WNXNZ(I)= WNX(I)-WNZ(I) LLEX1480
WNYNX(I)= WNY(I)-WNX(I) LLEX1490
W2GM(I) = OMEGA(I)*OMEGA(I)*GM(I) LLEX1500
PYNZ2(I)= 2.*( PY(I)- PZ(I) ) LLEX1510
PZNX2(I)= 2.*( PZ(I)- PX(I) ) LLEX1520
100 PXNY2(I)= 2.*( PX(I)- PY(I) ) LLEX1530
C      SET MODAL ANALYSIS MASS TO ZERO LLEX1540
DO 105 I = 1, 3 LLEX1550
105 AME1(I) = 0.0 LLEX1560
110 CONTINUE LLEX1570
C LLEX1580
C      AT PRESENT ALL FOOT PADS ARE OF THE SAME MASS LLEX1590
C LLEX1600
DO 70 I = 1, NFTPDS LLEX1610
70 AMFP(I) = FPMASS LLEX1620
C      INDFAR=3 IF ROTATIONS ARE NOT INTEGRATED LLEX1630
INDFAR = INDFXR+INDFYR+INDFZR LLEX1640

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C          PRECALCULATE MOMENT OF INERTIA CONSTANTS          LLEX1650
C  CIZZYY = CBIZZ-CBIYY          LLEX1660
C  CIXZZZ = CBIXX-CBIZZ          LLEX1670
C  CIYXX = CBIYY-CBIXX          LLEX1680
C          LLEX1690
C          SET INTEGRATION VARIABLE LIST AND CUTOFF LIST COUNTERS  LLEX1700
C          LLEX1710
C  IPTATL = 0          LLEX1720
C  IPTOTL = 0          LLEX1730
C          INITIALIZE THE INTEGRATION ROUTINES          LLEX1740
C  CALL SETUP          LLEX1750
C          LLEX1760
C          XS, XSD, AND XSDD          ARE INTEGRATION V. LLEX1770
C  IF ( INDFXD .NE. 0 ) GO TO 210          LLEX1780
C  CALL INUPD ( 364 )          LLEX1790
C  CALL INUPD ( 4 )          LLEX1800
C  CALL INUPD ( 8 )          LLEX1810
C 210 IF ( INDFYD .NE. 0 ) GO TO 220          LLEX1820
C          YS, YSD, AND YSDD          ARE INTEGRATION V. LLEX1830
C  CALL INUPD ( 12 )          LLEX1840
C  CALL INUPD ( 16 )          LLEX1850
C  CALL INUPD ( 20 )          LLEX1860
C 220 IF ( INDFZD .NE. 0 ) GO TO 230          LLEX1870
C          ZS, ZSD, AND ZSDD          ARE INTEGRATION V. LLEX1880
C  CALL INUPD ( 24 )          LLEX1890
C  CALL INUPD ( 28 )          LLEX1900
C  CALL INUPD ( 32 )          LLEX1910
C 230 IF ( INDFXR .NE. 0 ) GO TO 240          LLEX1920
C          WX, WXD, PHID, AND PHI          ARE INTEGRATION V. LLEX1930
C  CALL INUPD ( 44 )          LLEX1940
C  CALL INUPD ( 48 )          LLEX1950
C  CALL INUPD ( 40 )          LLEX1960
C  CALL INUPD ( 36 )          LLEX1970
C 240 IF ( INDFYR .NE. 0 ) GO TO 250          LLEX1980
C          WY, WYD, THTAD, AND THTA          ARE INTEGRATION V. LLEX1990
C  CALL INUPD ( 60 )          LLEX2000
C  CALL INUPD ( 64 )          LLEX2010
C  CALL INUPD ( 56 )          LLEX2020
C  CALL INUPD ( 52 )          LLEX2030
C 250 IF ( INDFZR .NE. 0 ) GO TO 260          LLEX2040
C          WZ, WZD, PSID, AND PSI          ARE INTEGRATION V. LLEX2050
C  CALL INUPD ( 76 )          LLEX2060
C  CALL INUPD ( 80 )          LLEX2070
C  CALL INUPD ( 72 )          LLEX2080
C  CALL INUPD ( 68 )          LLEX2090
C 260 CONTINUE          LLEX2100
C          GC(I), GCD(I), AND GCDD(I)          ARE INTEGRAT. V. LLEX2110
C  IF ( NMODES .EQ. 0 ) GO TO 290          LLEX2120
C  DO 280 I = 1, NMODES          LLEX2130
C  J = 4*I          LLEX2140
C  CALL INUPD ( 80+ J )          LLEX2150
C  CALL INUPD ( 100+ J )          LLEX2160
C 280 CALL INUPD ( 120+ J )          LLEX2170
C 290 CONTINUE          LLEX2180
C 1 CONTINUE          LLEX2190
C          LLEX2200

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C          CALL GEOM
C
C          IF ( INDIFP .EQ. 1) GO TO 64
C
C          INITIALIZE FOOT PAD POSITIONS IN SURFACE COORDINATES
C          ALL SO SET UP FOOT PAD CUTOFF VARIABLES AND
C          INTEGRATION QUANTIES
C
C          XS = 100.*GRAVE
C          YS = 0.0
C          ZS = 0.0
C          DO 60 I = 1, NFTPDS
C          A = DC(1,1)*XFP(I) + DC(1,2)*YFP(I) + DC(1,3)*ZFP(I)
C          1 - ATTH(I)
C          60 XS = AMIN1( XS, A )
C          XS = -XS
C
C          PRINT LANDER INITIAL CONDITONS
C
C          WRITE(6,5010)XS,YS,ZS,XSD,YSD,ZSD,WXIN,WYIN,WZIN
5010 FORMAT(1H1,/20X,46HLANDER INITIAL CONDITIONS (SURFACE COORDINATE
*       7HSYSTEM),//37X,1H(,10X,1HY,
*       10X,1HZ//6X,21HINITIAL C.G. POSITION,6X,3(F9.2,2X),//
*       6X,21HINITIAL C.G. VELOCITY,6X,3(F9.2,2X)//
*       6X,24HINITIAL ANGULAR VELOCITY,3X,3(F9.2,2X))
C          DO 62 I = 1, NFTPDS
C          XFPS(1,I) = XS+DC(1,1)*XFP(I)+DC(1,2)*YFP(I)+DC(1,3)*ZFP(I)
C          J = 4*I
C          IF ( XFPS(1,I)-ATTH(I) .LT. CUTERR ) GO TO 61
C          XFPS( I ) IS A CUTOFF VARIABLE
C          CALL LOC(140+J,ATTH(I)+(.5+GRAVE*2.5E-06)*CUTERR,1)
C          INDFPI( I ) = IPTATL
C          INDFPC(I) = 0
C          GO TO 62
C          XFPS(I),YFPS(I),ZFPS(I),XFPSD(I),YFPSD(I), ZFPSD(I)
C          XFPSDD(I), YFPSDD(I), ZFPSDD(I) ARE INTEGRATION VAR.
61 CALL INUPD ( 140 + J )
C          INDFPI( I ) = IPTOTL
C          CALL INUPD ( 160 + J )
C          CALL INUPD ( 180 + J )
C          CALL INUPD ( 200 + J )
C          CALL INUPD ( 220 + J )
C          CALL INUPD ( 240 + J )
C          CALL INUPD ( 260 + J )
C          CALL INUPD ( 280 + J )
C          CALL INUPD ( 300 + J )
C          YFPS(1,I) = YS+DC(2,1)*XFP(I)+DC(2,2)*YFP(I)+DC(2,3)*ZFP(I)
C          ZFPS(1,I) = ZS+DC(3,1)*XFP(I)+DC(3,2)*YFP(I)+DC(3,3)*ZFP(I)
C          A = WY * ZFP(I) - WZ * YFP(I)
C          B = WZ * XFP(I) - WX * ZFP(I)
C          C = WX * YFP(I) - WY * XFP(I)
C          XFPSD(1,I) = XSD + DC(1,1)*A + DC(1,2)*B + DC(1,3)*C
C          YFPSD(1,I) = YSD + DC(2,1)*A + DC(2,2)*B + DC(2,3)*C
C          ZFPSD(1,I) = ZSD + DC(3,1)*A + DC(3,2)*B + DC(3,3)*C
C          SET THIS FOOT PAD*S CONTACT INDICATOR
C          INDFPC ( I ) = 2

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C	62 CONTINUE	LLEX2770
C		LLEX2780
C	SAVE NON INTEGRATION TIMEHISTORY VARIABLES IN CASE	LLEX2790
C	OF INITIAL BACKUP	LLEX2800
	DO 63 I = 1, LTIMHS	LLEX2810
	63 TIMHSC(I) = TIMHSA(I)	LLEX2820
	64 CONTINUE	LLEX2830
C		LLEX2840
C	PROCESS THE FT. PAD. CALLS	LLEX2850
C		LLEX2860
	DO 380 I = 1, 6	LLEX2870
380	FLTL(I) = 0.0	LLEX2880
	IF (NMODES .EQ. 0) GO TO 395	LLEX2890
	DO 390 I = 1, 3	LLEX2900
	AME2(I) = 0.0	LLEX2910
390	GF(I) = 0.0	LLEX2920
395	CONTINUE	LLEX2930
	DO 430 IFP = 1, NFTPDS	LLEX2940
	IF (INDFPC(IFP) .LE. 0) GO TO 405	LLEX2950
C		LLEX2960
C	CONTACTING FOOTPAD	LLEX2970
C		LLEX2980
	IF(INDIFP.EQ.1) CALL FTPAD	LLEX2990
C		LLEX3000
	IF(IBOTM.NE.0)GO TO 3110	LLEX3010
C	SUM UP THE RESULTAN LEG FORCES	LLEX3020
	FLTL(1) = FLTL(1) + FLXS	LLEX3030
	FLTL(2) = FLTL(2) + FLYS	LLEX3040
	FLTL(3) = FLTL(3) + FLZS	LLEX3050
	FLTL(4) = FLTL(4) + TLXL	LLEX3060
	FLTL(5) = FLTL(5) + TLYL	LLEX3070
	FLTL(6) = FLTL(6) + TLZL	LLEX3080
	IF (NMODES .EQ. 0) GO TO 406	LLEX3090
C	SUM THE GEN. FORCES TO ELASTIC EFFECTS OF THE MAIN AND	LLEX3100
C	DRAG STRUTS	LLEX3110
	DO 404 I = 1, NMODES	LLEX3120
404	GF(I) = GF(I) + GFLEGS(I)	LLEX3130
406	CONTINUE	LLEX3140
C	F.P. ACCEL. =(GRAVITY LOADS -STRUT LOADS+STOIL LOADS)/F.P. MASS	LLEX3150
	XFPSDD(1,IFP) =(-GCOSZT*AMFP(IFP)+FSXSI(IFP)+SOILX(IFP))/AMFP(IFP)	LLEX3160
	YFPSDD(1,IFP) =(LLEX3170
	+FSYSI(IFP)+SOILY(IFP))/AMFP(IFP)	LLEX3180
	ZFPSDD(1,IFP) =(G SINZT*AMFP(IFP)+FSZSI(IFP)+SOILZ(IFP))/AMFP(IFP)	LLEX3190
C	INTEGRATE TO GET XFPS, YFPS, AND ZFPS	LLEX3200
	CALL INTEG (XFPSDD(1,IFP), XFPSD(1,IFP))	LLEX3210
	CALL INTEG (XFPSD (1,IFP), XFPS (1,IFP))	LLEX3220
	CALL INTEG (YFPSDD(1,IFP), YFPSD(1,IFP))	LLEX3230
	CALL INTEG (YFPSD (1,IFP), YFPS (1,IFP))	LLEX3240
	CALL INTEG (ZFPSDD(1,IFP), ZFPSD(1,IFP))	LLEX3250
	CALL INTEG (ZFPSD (1,IFP), ZFPS (1,IFP))	LLEX3260
	GO TO 430	LLEX3270
C		LLEX3280
C	NONCONTACTING FOOTPAD	LLEX3290
C		LLEX3300
405	CONTINUE	LLEX3310
	XXX=XFP(IFP)	LLEX3320
	YYY=YFP(IFP)	LLEX3330

	ZZZ=ZFP(IFP)	LLEX3330
	IF(NMODES.EQ.0)GO TO 320	LLEX3340
	IF(ILEG.EQ.0)GO TO 304	LLEX3350
	RRRT=DRAGST/SLNGMS(IFP)	LLEX3360
	XXX=XMSCB(IFP)+RRRT*(XFP(IFP)-XMSCB(IFP))	LLEX3370
	YYY=YMSCB(IFP)+RRRT*(YFP(IFP)-YMSCB(IFP))	LLEX3380
	ZZZ=ZMSCB(IFP)+RRRT*(ZFP(IFP)-ZMSCB(IFP))	LLEX3390
304	CONTINUE	LLEX3400
	XMST=XMSCB(IFP)	LLEX3410
	YMST=YMSCB(IFP)	LLEX3420
	ZMST=ZMSCB(IFP)	LLEX3430
	NN1=2*(IFP-1)+1	LLEX3440
	NN2=NN1+1	LLEX3450
	XDS1T=XDSCB(NN1)	LLEX3460
	YDS1T=YDSCB(NN1)	LLEX3470
	ZDS1T=ZDSCB(NN1)	LLEX3480
	XDS2T=XDSCB(NN2)	LLEX3490
	YDS2T=YDSCB(NN2)	LLEX3500
	ZDS2T=ZDSCB(NN2)	LLEX3510
	DO 305 I=1,NMODES	LLEX3520
	XMST=XMST+PMSX(IFP,I)*GC(1,I)	LLEX3530
	YMST=YMST+PMSY(IFP,I)*GC(1,I)	LLEX3540
	ZMST=ZMST+PMSZ(IFP,I)*GC(1,I)	LLEX3550
	XDS1T=XDS1T+PDSX(NN1,I)*GC(1,I)	LLEX3560
	YDS1T=YDS1T+PDSY(NN1,I)*GC(1,I)	LLEX3570
	ZDS1T=ZDS1T+PDSZ(NN1,I)*GC(1,I)	LLEX3580
	XDS2T=XDS2T+PDSX(NN2,I)*GC(1,I)	LLEX3590
	YDS2T=YDS2T+PDSY(NN2,I)*GC(1,I)	LLEX3600
	ZDS2T=ZDS2T+PDSZ(NN2,I)*GC(1,I)	LLEX3610
305	CONTINUE	LLEX3620
310	CONTINUE	LLEX3630
	DXMS=XXX-XMST	LLEX3640
	DYMS=YYY-YMST	LLEX3650
	DZMS=ZZZ-ZMST	LLEX3660
	DXDS1=XXX-XDS1T	LLEX3670
	DYDS1=YYY-YDS1T	LLEX3680
	DZDS1=ZZZ-ZDS1T	LLEX3690
	DXDS2=XXX-XDS2T	LLEX3700
	DYDS2=YYY-YDS2T	LLEX3710
	DZDS2=ZZZ-ZDS2T	LLEX3720
	FUNMS = DXMS**2+DYMS**2+DZMS**2-SLNGMS(IFP)**2	LLEX3730
	IF(ILEG.NE.0)FUNMS=DXMS**2+DYMS**2+DZMS**2-DRAGST**2	LLEX3740
	FUNDS1=DXDS1**2+DYDS1**2+DZDS1**2-SLNGDS(NN1)**2	LLEX3750
	FUNDS2=DXDS2**2+DYDS2**2+DZDS2**2-SLNGDS(NN2)**2	LLEX3760
	ERR=ABS(FUNMS)+ABS(FUNDS1)+ABS(FUNDS2)	LLEX3770
	IF(ERR.LT.(.00005*GRAVE)) GO TO 320	LLEX3780
	AAA=4.*(DYDS1*DZDS2-DZDS1*DYDS2)	LLEX3790
	BBB=4.*(DZDS1*DXDS2-DXDS1*DZDS2)	LLEX3800
	CCC=4.*(DYDS2*DXDS1-DXDS2*DYDS1)	LLEX3810
	DELL=2.*(DXMS*AAA+DYMS*BBB+DZMS*CCC)	LLEX3820
	DELX=-FUNMS*AAA-FUNDS1*BBB-FUNDS2*CCC	LLEX3830
	DELY=4.*DXMS*(DZDS1*FUNDS2-DZDS2*FUNDS1)-FUNMS*BBB+4.*DZMS*	LLEX3840
	*(DXDS2*FUNDS1-DXDS1*FUNDS2)	LLEX3850
	DELZ=4.*DXMS*(DYDS2*FUNDS1-DYDS1*FUNDS2)+4.*DYMS*(DXDS1*FUNDS2-	LLEX3860
	*DXDS2*FUNDS1)-FUNMS*CCC	LLEX3870
	XXX=XXX+DELX/DELL	LLEX3880

	YYY=YYY+DELY/DELL	LLEX3890
	ZZZ=ZZZ+DELZ/DELL	LLEX3900
	GO TO 310	LLEX3910
320	CONTINUE	LLEX3920
	IF(ILEG.EQ.0)GO TO 321	LLEX3930
	IF(NMODES.EQ.0)GO TO 321	LLEX3935
	CX=(XXX-XMSCB(IPF))/DRAGST	LLEX3940
	CY=(YYY-YMSCB(IPF))/DRAGST	LLEX3950
	CZ=(ZZZ-ZMSCB(IPF))/DRAGST	LLEX3960
	DDD=SLNGMS(IPF)-DRAGST	LLEX3970
	XXX=XXX+CX*DDD	LLEX3980
	YYY=YYY+CY*DDD	LLEX3990
	ZZZ=ZZZ+CZ*DDD	LLEX4000
321	CONTINUE	LLEX4010
	XFPS(1,IFP)=XS+DC(1,1)*XXX+DC(1,2)*YYY+DC(1,3)*ZZZ	LLEX4020
	YFPS(1,IFP)=YS+DC(2,1)*XXX+DC(2,2)*YYY+DC(2,3)*ZZZ	LLEX4030
	ZFPS(1,IFP)=ZS+DC(3,1)*XXX+DC(3,2)*YYY+DC(3,3)*ZZZ	LLEX4040
	NNN=2*(IFP-1)+1	LLEX4050
	FORMS(IFP)=0.0	LLEX4060
	FORDS(NNN)=0.0	LLEX4070
	FORDS(NNN+1)=0.0	LLEX4080
	IF(NFORC.EQ.0) GO TO 430	LLEX4090
	SAVMSX(IFP)=0.0	LLEX4100
	SAVMSY(IFP)=0.0	LLEX4110
	SAVMSZ(IFP)=0.0	LLEX4120
	SAVDSX(NNN)=0.0	LLEX4130
	SAVDSX(NNN+1)=0.0	LLEX4140
	SAVDSY(NNN)=0.0	LLEX4150
	SAVDSY(NNN+1)=0.0	LLEX4160
	SAVDSZ(NNN)=0.0	LLEX4170
	SAVDSZ(NNN+1)=0.0	LLEX4180
430	CONTINUE	LLEX4190
	INDIFP = 1	LLEX4200
C		LLEX4210
C	SET INDFPD TO INDICATE THE HIGHEST FOOTPAD STATUS	LLEX4220
C		LLEX4230
	INDFPD = INDFPC(1)	LLEX4240
	DO 435 IFP = 2,6	LLEX4250
435	INDFPD = MIN0(INDFPD, INDFPC(IFP))	LLEX4260
C		LLEX4270
C	UPDATE THE MASS MATRIX WITH THE ELASTIC MODE	LLEX4280
C	CONTRIBUTIONS	LLEX4290
	IF (NMODES .EQ. 0) GO TO 480	LLEX4300
	DO 450 I = 1,NMODES	LLEX4310
	AME2(1)=AME2(1) + (PYZ2(I)+WNX(I)*GC(1,I))*GC(1,I)	LLEX4320
	AME2(2)=AME2(2) + (PXZ2(I)+WNY(I)*GC(1,I))*GC(1,I)	LLEX4330
450	AME2(3)=AME2(3) + (PXY2(I)+WNZ(I)*GC(1,I))*GC(1,I)	LLEX4340
	DO 470 I = 1, 3	LLEX4350
	J = I + 3	LLEX4360
470	AM(J,J) = AM(J,J) - (AME1(I) - AME2(I))	LLEX4370
480	CONTINUE	LLEX4380
	IF (INDFPD .GT. 1) GO TO 540	LLEX4390
C		LLEX4400
C	PROCESS THE NONCONTACTING FOOTPAD MASS CONTRIBUTIONS	LLEX4410
C		LLEX4420
	DO 490 I = 1, 3	LLEX4430

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DO 490 J = 1, 3                                LLEX4440
K = J+3                                        LLEX4450
AM(I,K) = 0.0                                  LLEX4460
490 AM(K,I) = 0.0                              LLEX4470
DO 530 IFP=1,NFTPDS                            LLEX4480
IF ( INDFPC(IFP) .GT. 0 ) GO TO 520           LLEX4490
DO 500 I = 1, 3                                LLEX4500
C          SET UPPER-RIGHT HAND CORNER OF THE MASS MATRIX LLEX4510
AM(I,4)=AM(I,4)+ AMFP(IFP)*(-DC(I,2)*ZFP(IFP)+DC(I,3)*YFP(IFP) ) LLEX4520
AM(I,5)=AM(I,5)+ AMFP(IFP)*( DC(I,1)*ZFP(IFP)-DC(I,3)*XFP(IFP) ) LLEX4530
500 AM(I,6)=AM(I,6)+ AMFP(IFP)*(-DC(I,1)*YFP(IFP)+DC(I,2)*XFP(IFP) ) LLEX4540
C          SET LOWER-LEFT HAND CORNER OF THE MASS MATRIX LLEX4550
AM(5,1) = AM(5,1) + AMFP(IFP) * ZFPS(1,IFP) LLEX4560
AM(6,1) = AM(6,1) - AMFP(IFP) * YFPS(1,IFP) LLEX4570
AM(6,2) = AM(6,2) + AMFP(IFP) * XFPS(1,IFP) LLEX4580
IF (INDFPC(IFP).LT. 0 ) GO TO 530             LLEX4590
INDFPC(IFP) = -1                              LLEX4600
AK = AMFP(IFP)                                LLEX4610
510 A = YFP(IFP)*YFP(IFP)                    LLEX4620
B = ZFP(IFP)*ZFP(IFP)                        LLEX4630
C = XFP(IFP)*XFP(IFP)                        LLEX4640
AM(1,1) = AM(1,1) + AK                        LLEX4650
AM(4,4) = AM(4,4) + AK*( A + B )             LLEX4660
AM(5,5) = AM(5,5) + AK*( C + B )             LLEX4670
AM(6,6) = AM(6,6) + AK*( C + A )             LLEX4680
AM(5,4) = AM(5,4) - AK* XFP(IFP) * YFP(IFP) LLEX4690
AM(6,4) = AM(6,4) - AK* XFP(IFP) * ZFP(IFP) LLEX4700
AM(6,5) = AM(6,5) - AK* YFP(IFP) * ZFP(IFP) LLEX4710
GO TO 530                                     LLEX4720
520 IF ( INDFPC(IFP) .GT. 1 ) GO TO 530       LLEX4730
AK = -AMFP(IFP)                              LLEX4740
INDFPC(IFP) = 2                              LLEX4750
GO TO 510                                     LLEX4760
530 CONTINUE                                  LLEX4770
AM(2,2) = AM(1,1)                            LLEX4780
AM(3,3) = AM(1,1)                            LLEX4790
AM(4,5) = AM(5,4)                            LLEX4800
AM(4,6) = AM(6,4)                            LLEX4810
AM(5,6) = AM(6,5)                            LLEX4820
AM(4,2) = -AM(5,1)                           LLEX4830
AM(4,3) = -AM(6,1)                           LLEX4840
AM(5,3) = -AM(6,2)                           LLEX4850
C                                               LLEX4860
C          IF NECESSARY, DEFIND THE MASS MATRIX INVERSE LLEX4870
C                                               LLEX4880
C          CALL MATINV( AM(1,1), AMINV(1,1), 6 ) LLEX4890
540 CONTINUE                                  LLEX4900
C                                               LLEX4910
C          SET UP THE FORCE/TORQUE ARRAY LLEX4920
C                                               LLEX4930
C          LEG FORCE/TORQUE EFFECTS LLEX4940
DO 600 I = 1, 6                                LLEX4950
600 FTS(I) =FLTL(I)                            LLEX4960
C          PLANET GRAVITY EFFECTS LLEX4970
FTS(1) = FTS(1) - GCOSZT* AM(1,1)           LLEX4980
FTS(3) = FTS(3) + GSINZT* AM(1,1)           LLEX4990

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A   = WX * WX
B   = WY * WY
C   = WZ * WZ
IF ( INDFAR .EQ. 3 ) GO TO 610
FO(1) = -WX*(WZ*CBIXY-WY*CBIXZ)-(C-B)*CBIYZ-WY*WZ*CIZZZY
FO(2) = -WX*(WX*CBIZY-WZ*CBIXY)-(A-C)*CBIXZ-WX*WZ*CIXXZZ
FO(3) = -WZ*(WY*CBIXZ-WX*CBIZY)-(B-A)*CBIXY-WX*WY*CIYXX
C   RIGID BODY INERTIA LOAD EFFECTS
FTS(4) = FTS(4)+FO(1)
FTS(5) = FTS(5)+FO(2)
FTS(6) = FTS(6)+FO(3)
610 CONTINUE
IF ( NMODES .EQ. 0 ) GO TO 640
C   INERTIA LOADS DUE TO ELASTIC MODES
DO 625 I = 1,6
625 FE(I) = 0.0
DO 630 I = 1, NMODES
FE(1) = FE(1)+ ( PYZ(I)+WNX(I)*GC(1,I) ) *GCD(1,I)
FE(2) = FE(2)+ ( PXZ(I)+WNY(I)*GC(1,I) ) *GCD(1,I)
FE(3) = FE(3)+ ( PXY(I)+WNZ(I)*GC(1,I) ) *GCD(1,I)
FE(4) = FE(4)+ ( PYNZ2(I)+ WNZNY(I)*GC(1,I) ) *GC (1,I)
FE(5) = FE(5)+ ( PZNX2(I)+ WNXNZ(I)*GC(1,I) ) *GC (1,I)
630 FE(6) = FE(6)+ ( PXNY2(I)+ WNYNX(I)*GC(1,I) ) *GC (1,I)
FE(4) = FE(4) + WX*( FE(1) + FE(1) )
FE(5) = FE(5) + WY*( FE(2) + FE(2) )
FE(6) = FE(6) + WZ*( FE(3) + FE(3) )
FTS(4) = FTS(4) - FE(4)
FTS(5) = FTS(5) - FE(5)
FTS(6) = FTS(6) - FE(6)
640 CONTINUE
C
C   CALCULATE THE CENTER BODY ACCELERATIONS AND ANGULAR RATES OF
C   CHANGE
DO 670 I= 1, 6
AHOLD(I) = 0.0
DO 670 J= 1, 6
670 AHOLD(I) = AHOLD(I)+ AMINV(I,J)*FTS(J)
XSDD = AHOLD(1)
YSDD = AHOLD(2)
ZSDD = AHOLD(3)
WXD = AHOLD(4)
WYD = AHOLD(5)
WZD = AHOLD(6)
IF(INDFXD.NE.0)XSDD=0.0
IF(INDFYD.NE.0)YSDD=0.0
IF(INDFZD.NE.0)ZSDD=0.0
IF(INDFXR.NE.0)WXD=0.0
IF(INDFYR.NE.0)WYD=0.0
IF(INDFZR.NE.0)WZD=0.0
IF ( NMODES .EQ. 0 ) GO TO 750
C   DRAG STRUT
C   TOTAL GENERAL FORCES
C   ELASTIC-ROTATIONAL COUPLINGS (GFO)
C   MAIN STRUT
DO 730 I = 1, NMODES
GFO(I) = A*( PYZ(I)+WNX(I)*GC(1,I))+ B*(PXZ(I)+WNY(I)*GC(1,I))

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LLEX5000
LLEX5010
LLEX5020
LLEX5030
LLEX5040
LLEX5050
LLEX5060
LLEX5070
LLEX5080
LLEX5090
LLEX5100
LLEX5110
LLEX5120
LLEX5130
LLEX5140
LLEX5150
LLEX5160
LLEX5170
LLEX5180
LLEX5190
LLEX5200
LLEX5210
LLEX5220
LLEX5230
LLEX5240
LLEX5250
LLEX5260
LLEX5270
LLEX5280
LLEX5290
LLEX5300
LLEX5310
LLEX5320
LLEX5330
LLEX5340
LLEX5350
LLEX5360
LLEX5370
LLEX5380
LLEX5390
LLEX5400
LLEX5410
LLEX5420
LLEX5430
LLEX5440
LLEX5450
LLEX5460
LLEX5470
LLEX5480
LLEX5490
LLEX5500
LLEX5510
LLEX5520
LLEX5530
LLEX5540
LLEX5550

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1          GF(I) = GFO(I)+GF(I)          + C*(PXY(I)+WNZ(I)*GC(1,I))    LLEX5560
C          GF(I) = GFO(I)+GF(I)          LLEX5570
C          730 GCDD(1,I)=(GF(I)-W2GM(I)*GC(1,I) ) / GM(I)          LLEX5580
C          750 CONTINUE          LLEX5590
C          IF ( INDFXD .NE. 0 ) GO TO 1000          LLEX5600
C          CALL INTEG ( XSDD, XSD )          LLEX5610
C          CALL INTEG ( XSD , XS )          LLEX5620
1000      IF ( INDFYD .NE. 0 ) GO TO 1010          LLEX5630
C          CALL INTEG ( YSDD, YSD )          LLEX5640
C          CALL INTEG ( YSD , YS )          LLEX5650
1010      IF ( INDFZD .NE. 0 ) GO TO 1020          LLEX5660
C          CALL INTEG ( ZSDD, ZSD )          LLEX5670
C          CALL INTEG ( ZSD , ZS )          LLEX5680
1020      IF ( INDFXR .NE. 0 ) GO TO 1030          LLEX5690
C          CALL INTEG ( WXD , WX )          LLEX5700
C          CALL INTEG ( PHID, PHI )          LLEX5710
1030      IF ( INDFYR .NE. 0 ) GO TO 1040          LLEX5720
C          CALL INTEG ( WYD , WY )          LLEX5730
C          CALL INTEG ( THTAD, THTA )          LLEX5740
1040      IF ( INDFZR .NE. 0 ) GO TO 1050          LLEX5750
C          CALL INTEG ( WZD , WZ )          LLEX5760
C          CALL INTEG ( PSID, PSI )          LLEX5770
1050      CONTINUE          LLEX5780
C          IF ( NMODES .EQ. 0 ) GO TO 1070          LLEX5790
C          DO 1060 I= 1, NMODES          LLEX5800
C          CALL INTEG ( GCDD(1,I), GCD(1,I) )          LLEX5810
1060      CALL INTEG ( GCD (1,I), GC (1,I) )          LLEX5820
1070      CONTINUE          LLEX5830
C          *****          LLEX5840
C          IF ( IPTATL .EQ. 0 ) GO TO 2010          LLEX5850
C          IF ( IPTATL .EQ. 0 ) GO TO 2010          LLEX5860
C          CHECK FOR A VARIABLE PAST OR AT IT*S CUTOFF VALUE.          LLEX5870
C          CALL CUT          LLEX5880
C          JCUT VARIABLE AT CUTOFF          LLEX5890
C          IF ( JCUT .GT. 0 ) GO TO 3000          LLEX5900
C          IF ( JCUT .EQ. 0 ) GO TO 2010          LLEX5910
C          JCUT VARIABLE PAST IT*S CUTOFF VALUE          LLEX5920
C          DO 2005 I = 1,45          LLEX5930
2005      TIMHSA(I) = TIMHSC(I)          LLEX5940
C          DO 2006 I = 52, LTIMHS          LLEX5950
2006      TIMHSA(I) = TIMHSC(I)          LLEX5960
C          INDBPU = 1          LLEX5970
C          RESET FOOTPAD CONTACT INDICATOR IF THEY WERE JUST CHANGED          LLEX5980
C          AFTER THE CALL TO UPDATE.          LLEX5990
C          IF ( INDIVR .EQ. 0 ) GO TO 1          LLEX6000
C          DO 2007 I = 1, INDIVR          LLEX6010
C          IFP = INOUTV(I)          LLEX6020
2007      INDFPC(IFP) = 2          LLEX6030
C          INDIVR = 0          LLEX6040
C          GO TO 1          LLEX6050
2010      CONTINUE          LLEX6060
C          *****          LLEX6070
C          *****          LLEX6080
C          *****          LLEX6090
C          *****          LLEX6100
C          *****          LLEX6110

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	IF (IFIN .NE. 0) GO TO 2000	LLEX6120
	IF (IVARH .EQ. 1) GO TO 1910	LLEX6130
	IF (IVAL .LT. 0) GO TO 2000	LLEX6140
1910	IF (INDIRV .EQ. 0) GO TO 1920	LLEX6150
	DO 424 IK= 1, INDIRV	LLEX6160
	IFP = INOUTV(IK)	LLEX6170
C	THIS FOOTPAD HAS JUST LEFT THE LANDING SURFACE. REMOVE	LLEX6180
C	ITS DISPLACEMENT, VELOCITY, AND ACCELATION VARIABLES	LLEX6190
C	FROM THE INTEGRATION LIST	LLEX6200
	J = INDFPI(IFP)	LLEX6210
	K = J+9	LLEX6220
	IF (K .GT. IPTOTL) GO TO 421	LLEX6230
	DO 420 I = K, IPTOTL	LLEX6240
	LOCNAM(J) = LOCNAM(I)	LLEX6250
420	J = J + 1	LLEX6260
421	CONTINUE	LLEX6270
	IPTOTL = IPTOTL - 9	LLEX6280
C	SET XFPS AS A CUTOFF VARIABLE	LLEX6290
	CALL LOC(140+4*IFP,ATTH(IFP)+(.5+GRAVE*2.5E-06)*CUTERR,1)	LLEX6300
C		LLEX6310
C	UPDATE THE INTEGRATION VARIABLE POINTERS	LLEX6320
C		LLEX6330
	DO 423 I = 1, NFTPDS	LLEX6340
	IF (INDFPC(I) .LE. 0) GO TO 423	LLEX6350
	IF (INDFPI(I).GT. INDFPI(IFP)) INDFPI(I) = INDFPI(I) - 9	LLEX6360
423	CONTINUE	LLEX6370
	IF (IK .EQ. INDIRV) GO TO 425	LLEX6380
	K = IK +1	LLEX6390
	DO 422 I = K, INDIRV	LLEX6400
	J = INOUTV(I)	LLEX6410
	IF (INDFPI(J) .GT. INDFPI(IFP)) INDFPI(J) = INDFPI(J) - 9	LLEX6420
422	CONTINUE	LLEX6430
425	CONTINUE	LLEX6440
C	UPDATE THE CUTOFF VARIABLE POINTER	LLEX6450
	INDFPI(IFP) = IPTATL	LLEX6460
424	CONTINUE	LLEX6470
C	*****	LLEX6480
1920	CONTINUE	LLEX6490
C	IPRNT .EQ. 0 EACH TIME THE VARIABLE (TIME) IS A MULTIPLE	LLEX6500
C	OF HMAX.	LLEX6510
	IF (IPRNT .NE. 0) GO TO 2000	LLEX6520
	KPTCNT = KPTCNT + 1	LLEX6530
	IF(IFPPRT.EQ.0)GO TO 1999	LLEX6540
	IPRINT=IPRTFP(1)	LLEX6550
	DO 1998 I=1,NFTPDS	LLEX6560
1998	IF(IPRTFP(I).LT.IPRINT)IPRINT=IPRTFP(I)	LLEX6570
1999	CONTINUE	LLEX6580
	IF(IFPPRT.EQ.0)IPRINT=IPTCNT	LLEX6590
	IF(KPTCNT.LT.IPRINT)GO TO 2000	LLEX6600
	KPTCNT = 0	LLEX6610
	TIMTST = TIME	LLEX6620
C		LLEX6630
	CALL OUTPUT	LLEX6640
	IF (HZ .LT. 1.E-32) STOP	LLEX6650
C		LLEX6660
C	CHECK FOR TIME MAX.	LLEX6670

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C      IF ( TIME.GE. TIMAX ) GO TO 3110                                LLEX6680
C      CHECK FOR STABILITY                                            LLEX6690
C      IF(ISTAB.NE.0)GO TO 3110                                       LLEX6700
C      *****LLEX6710
2000 CONTINUE                                                         LLEX6720
C      IF ( IFIN .NE. 0 ) GO TO 2060                                   LLEX6730
C      IF ( IVARH .EQ. 1 ) GO TO 2060                                   LLEX6740
C      IF ( IVAL .GE. 0 ) GO TO 2060                                   LLEX6750
C      IF THE INTEGRATION IS GOING TO BE BACKED UP TO REDUCE THE    LLEX6760
C      STEP SIZE, THEN THE TIME HISTORY COMMON MUST BE BACKED      LLEX6770
C      UP AT THIS TIME                                               LLEX6780
2020 CONTINUE                                                         LLEX6790
C      DO 2030 I = 1,45                                               LLEX6800
2030 TIMHSA(I) = TIMHSC(I)                                           LLEX6810
C      DO 2031 I = 52, LTIMHS                                         LLEX6820
2031 TIMHSA(I) = TIMHSC(I)                                           LLEX6830
C      INDBPU = 1                                                     LLEX6840
C      RESET FOOTPAD CONTACT INDICATOR IF THEY WERE JUST CHANGED    LLEX6850
C      AFTER THE CALL TO UPDATE.                                       LLEX6860
C      IF ( INDIVR .EQ. 0 ) GO TO 2060                                   LLEX6870
C      DO 2037 I = 1, INDIVR                                           LLEX6880
C      IFP = INOUTV(I)                                                 LLEX6890
2037 INDFPC(IFP) = 2                                                  LLEX6900
C      INDIVR = 0                                                     LLEX6910
C      LLEX6920
C      CALL INTEGRATION UPDATE                                         LLEX6930
C      LLEX6940
C      *****LLEX6950
2060 CALL UPDAT                                                       LLEX6960
C      IF ( IFIN .NE. 0 ) GO TO 1                                       LLEX6970
C      INDIVR=0                                                         LLEX6980
C      DO 2070 IFP = 1, NFTPDS                                         LLEX6990
C      IF ( INDFPC(IFP) .LE. 0 ) GO TO 2070                             LLEX7000
C      CHECK TO SEE IF FOOT PAD HAS LEFT THE GROUND                  LLEX7010
C      IF(XFPS(1,IFP).LT.ATTH(IFP)+2.*CUTERR) GO TO 2070            LLEX7020
C      LLEX7030
C      SAVE AND UPDATE INDICATORS FOR INTEGRATION VARIABLES TO BE    LLEX7040
C      REMOVE FROM THE INTEGRATION LIST IF THIS STEP                 LLEX7050
C      IS ACCEPT BY THE CUTOFF ROUTINE AND THE VARIABLE STEP        LLEX7060
C      SIZE OPTION                                                    LLEX7070
C      LLEX7080
C      INDIVR = INDIVR + 1                                             LLEX7090
C      INOUTV(INDIVR) = IFP                                           LLEX7100
C      SET THE CONTACT INDICATOR                                       LLEX7110
C      INDFPC(IFP)= 0                                                  LLEX7120
C      RECALCULATE THE F.P. LANDER COORDINATES                        LLEX7130
C      LANDER COORDINATES = DC**(-1)*(SURFACE COORDINATES F.P.     LLEX7140
C      - SURFACE COORDINATES C.G.) LLEX7150
C      A = XFPS(1,IFP) - XS                                           LLEX7160
C      B = YFPS(1,IFP) - YS                                           LLEX7170
C      C = ZFPS(1,IFP) - ZS                                           LLEX7180
C      XFP(IFP)=DC(1,1)*A + DC(2,1)*B + DC(3,1)*C                    LLEX7190
C      YFP(IFP)=DC(1,2)*A + DC(2,2)*B + DC(3,2)*C                    LLEX7200
C      ZFP(IFP)=DC(1,3)*A + DC(2,3)*B + DC(3,3)*C                    LLEX7210
2070 CONTINUE                                                         LLEX7220
C      IF ( NMODES .EQ. 0 ) GO TO 2090                                LLEX7230

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DO 2080 I = 1, 3                                LLEX7240
2080 AME1(I) = AME2(I)                            LLEX7250
2090 CONTINUE                                    LLEX7260
      DO 2095 I = 1, LTIMHS                       LLEX7270
      IF ( INDBPU .EQ. 1 ) GO TO 2095             LLEX7280
      TIMHSC(I) = TIMHSB(I)                       LLEX7290
2095 TIMHSB(I) = TIMHSA(I)                       LLEX7300
      INDBPU = 0                                   LLEX7310
      GO TO 1                                      LLEX7320
C          *****LLEX7330
C
C          MAKE SURE THIS TIME POINT IS PRINTED   LLEX7340
C
C          LLEX7350
C          LLEX7360
3000 IF ( TIMTST .NE. TIME ) CALL OUTPUT          LLEX7370
      TIMTST = TIME                               LLEX7380
      I =(IAD(JCUT) -140)                         LLEX7390
C          FIND THE SUBSCRIPT OF THE FOOT PAD    LLEX7400
C          ADD X-Y-ZFPS(I/4), X-Y-ZFPSD(I/4), AND X-Y-ZFPSDD(I/4) LLEX7410
C          TO INTEGRATION LIST                   LLEX7420
      CALL INUPD( 140 + I )                       LLEX7430
      J = I/4                                     LLEX7440
C
C          LLEX7450
C          UPDATE THE CUTOFF VARIABLE POINTERS    LLEX7460
C
C          LLEX7470
C          LLEX7480
DO 3005 K = 1, NFTPDS                            LLEX7480
      IF ( INDFPC(K) .GT. 0 ) GO TO 3005          LLEX7490
      IF ( INDFPI(K) .GT. INDFPI(J)) INDFPI(K) = INDFPI(K) - 1 LLEX7500
3005 CONTINUE                                    LLEX7510
C          UPDATE THE INTEGRATION VARIABLE POINTER LLEX7520
      INDFPI( J ) = IPTOTL                         LLEX7530
      CALL INUPD( 160 + I )                       LLEX7540
      CALL INUPD( 180 + I )                       LLEX7550
      CALL INUPD( 200 + I )                       LLEX7560
      CALL INUPD( 220 + I )                       LLEX7570
      CALL INUPD( 240 + I )                       LLEX7580
      CALL INUPD( 260 + I )                       LLEX7590
      CALL INUPD( 280 + I )                       LLEX7600
      CALL INUPD( 300 + I )                       LLEX7610
      A = WY * ZFP(J) - WZ * YFP(J)              LLEX7620
      B = WZ * XFP(J) - WX * ZFP(J)              LLEX7630
      C = WX * YFP(J) - WY * XFP(J)              LLEX7640
      XFPSD(1,J) = XSD + DC(1,1)*A + DC(1,2)*B + DC(1,3)*C LLEX7650
      YFPSD(1,J) = YSD + DC(2,1)*A + DC(2,2)*B + DC(2,3)*C LLEX7660
      ZFPSD(1,J) = ZSD + DC(3,1)*A + DC(3,2)*B + DC(3,3)*C LLEX7670
      INDFPC( J ) = 1                             LLEX7680
      I = JCUT + 1                                LLEX7690
      IF ( I .GT. IPTATL ) GO TO 3015             LLEX7700
      DO 3010 K= I ,IPTATL                        LLEX7710
      IAD(JCUT) = IAD(K)                          LLEX7720
      XSI(JCUT) = XSI(K)                          LLEX7730
      IND(JCUT) = IND(K)                          LLEX7740
      XMNI(JCUT) = XMNI(K)                        LLEX7750
3010 JCUT = JCUT + 1                              LLEX7760
3015 CONTINUE                                    LLEX7770
      IPTATL = IPTATL-1                           LLEX7780
C
C          LLEX7790

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C	CHECK FOR SIMULTANEOUS CUTOFF VARIABLES	LLEX7800
C		LLEX7810
	IF (IPTATL .EQ. 0) GO TO 3030	LLEX7820
	CALL CUT	LLEX7830
	IF (JCUT .GT. 0) GO TO 3000	LLEX7840
	IF (JCUT .LT. 0) GO TO 3200	LLEX7850
C		LLEX7860
C	RESET THE INTEGRATION PRINT AND STEP SIZE CONDITIONS	LLEX7870
C		LLEX7880
3030	CONTINUE	LLEX7890
	CALL SETUP	LLEX7900
	DO 3040 I = 1, 6	LLEX7910
3040	TIMHSA(I) = TIMHSB(I)	LLEX7920
	DO 3045 I = 43, 45	LLEX7930
3045	TIMHSA(I) = TIMHSB(I)	LLEX7940
	DO 3050 I = 58, LTIMHS	LLEX7950
3050	TIMHSA(I) = TIMHSB(I)	LLEX7960
	DO 3060 I = 1, LTIMHS	LLEX7970
3060	TIMHSC(I) = TIMHSA(I)	LLEX7980
	INDBPU = 1	LLEX7990
	GO TO 1	LLEX8000
C	*****	LLEX8010
C		LLEX8020
C	END OF CASE	LLEX8030
C		LLEX8040
3110	CONTINUE	LLEX8050
C		LLEX8060
	IF(IBOTM.NE.0)CALL OUTPUT	LLEX8070
C		LLEX8080
	CALL SUMMRY	LLEX8090
C		LLEX8100
	RETURN	LLEX8110
3200	CONTINUE	LLEX8120
	WRITE (6,3210)	LLEX8130
3210	FORMAT(52H TWO VARIABLES PASSED CUTOFF AT THE SAME TIME POINT,	LLEX8140
1	44H AND BOTH COULD NOT BE SUCCESSFULLY CUT BACK)	LLEX8150
	CALL OUTPUT	LLEX8160
9000	RETURN	LLEX8170
	END	LLEX8180

	SUBROUTINE FTPAD	FTPD 10
C		FTPD 20
C	THIS SUBROUTINE CONTAINS THE LOGIC FOR DETERMINING THE SOIL	FTPD 30
C	FORCES AND LANDING GEAR STRUT FORCES	FTPD 40
C		FTPD 50
C	FOLLOWING NOTATION USED -	FTPD 60
C	S.C.S. - SURFACE COORDINATE SYSTEM	FTPD 70
C	L.C.S. - LANDER COORDINATE SYSTEM	FTPD 80
C		FTPD 90
C	THE FOLLOWING INFORMATION IS RETURNED FOR THE LANDING GEAR UNDER	FTPD 100
C	CONSIDERATION.	FTPD 110
C	STRUT FORCES ON CENTER BODY IN S.C.S.	FTPD 120
C	STRUT FORCES ON FOOTPAD IN S.C.S.	FTPD 130
C	STRUT TORQUES ON CENTER BODY IN L.C.S.	FTPD 140
C	GENERALIZED FORCES	FTPD 150
C		FTPD 160
	DIMENSION XFPS(4,5), YFPS(4,5), ZFPS(4,5),	FTPD 170
1	XFPSD(4,5), YFPSD(4,5), ZFPSD(4,5),	FTPD 180
2	XFPSDD(4,5), YFPSDD(4,5), ZFPSDD(4,5)	FTPD 190
	DIMENSION GC(4,5), GCD(4,5), GCDD(4,5)	FTPD 200
	COMMON / THISV/ TIMHSA(1), ATTH(5),AM(6,6),AME1(3),INDFPI(6),	FTPD 210
1	INDFPC(6),	FTPD 220
2	STRPDS(10), STRPMS(5), IPOCDS(10), IPOCMS(5), URCDS (10),	FTPD 230
3	URTDS (10), URCMS (5), URTMS (5), SETCDS(10), SETTDS(10),	FTPD 240
4	SETCMS(5), SETTMS(5), INDCDS(10), INDTDS(10), INDCMS(10),	FTPD 250
5	INDTMS(5), PRFCDS (10), PRFTDS (10), PRFCMS (5), PRFTMS (5),	FTPD 260
6	IPRDS (10), IPRMS (5), FRVDSC(10), FRVDST(10), FRVMSC(5),	FTPD 270
L	FRVMST(5), IPOTDS(10), IPOTMS(5)	FTPD 280
	COMMON COMINT(400)	FTPD 290
	EQUIVALENCE (COMINT(3), IBOTM)	FTPD 300
	EQUIVALENCE (COMINT(4), XSD)	FTPD 310
	EQUIVALENCE (COMINT(12), YS)	FTPD 320
	EQUIVALENCE (COMINT(16), YSD)	FTPD 330
	EQUIVALENCE (COMINT(24), ZS)	FTPD 340
	EQUIVALENCE (COMINT(28), ZSD)	FTPD 350
	EQUIVALENCE (COMINT(44), WX)	FTPD 360
	EQUIVALENCE (COMINT(60), WY)	FTPD 370
	EQUIVALENCE (COMINT(76), WZ)	FTPD 380
	EQUIVALENCE (COMINT(84), GC)	FTPD 390
	EQUIVALENCE (COMINT(104), GCD)	FTPD 400
	EQUIVALENCE (COMINT(124), GCDD)	FTPD 410
	EQUIVALENCE (COMINT(144), XFPS)	FTPD 420
	EQUIVALENCE (COMINT(164), XFPSD)	FTPD 430
	EQUIVALENCE (COMINT(184), XFPSDD)	FTPD 440
	EQUIVALENCE (COMINT(204), YFPS)	FTPD 450
	EQUIVALENCE (COMINT(224), YFPSD)	FTPD 460
	EQUIVALENCE (COMINT(244), YFPSDD)	FTPD 470
	EQUIVALENCE (COMINT(264), ZFPS)	FTPD 480
	EQUIVALENCE (COMINT(284), ZFPSD)	FTPD 490
	EQUIVALENCE (COMINT(304), ZFPSDD)	FTPD 500
	EQUIVALENCE (COMINT(324), TIME)	FTPD 510
	EQUIVALENCE (COMINT(343), IFIN)	FTPD 520
	EQUIVALENCE (COMINT(361), IPTCNT)	FTPD 530
	EQUIVALENCE (COMINT(364), XS)	FTPD 540
C	COMINT(364-367) USED BY XS	FTPD 550

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EQUIVALENCE ( COMINT( 368 ), CURDT )
COMMON
1  CBMASS, CBIXX , CBIXZ , CBIYY , CBIYZ , CBIZZ , FPMASS,CBIXY,
2  DC(3,3) , XFP(5), YFP(5), ZFP(5), WNX(5), WNY(5),
3  WNZ(5), PX(5) , PY(5) , PZ(5) , GM(5) , OMEGA(5) ,
4  GRAV , GRAVE , ZETA , FTS (6) , FSXSI(5) ,
5  FSYSI(5) , FSZSI(5) , SOILX(5) ,
6  SOILY(5) , SOILZ(5) , PMSX(5,5) ,
7  PMSY(5,5) , PMSZ(5,5) , PDSX(10,5) ,
8  PDSY(10,5) , PDSZ(10,5) , FLXS , FLYS , FLZS ,
9  TLXL , TLYL , TLZL , SLO , XMSCB(5) ,
C  YMSCB(5) , ZMSCB(5) , XDSCB(10) ,
D  YDSCB(10) , ZDSCB(10) , ILEG , IMS ,
E  FSTX , FSTY , FSTZ , PVCBX , PVCBY , PVCBZ ,
F  PVFPX , PVFPY , PVFPZ , NOLEG , SLOMS(5) ,
G  SLODS(10)
COMMON
1  PFCMS(5) , PFCDS(5) , PFTDS(5) ,
2  PFTMS(5) , SRCDS(5) , COEFS(5) ,
3  SRTDS(5) , SRTMS(5) , COEFMS, COEFMS, GAMDS ,
4  GAMMS , SRUCDS, SRUCMS, SRUTDS, SRUTMS, SCMXDS, SCMXMS,
5  STMXDS, STMXMS, CDCDS(5) , CDCMS(5) ,
6  CDTDS(5) , CDMS(5) , FRICDS, FRICMS, IRETDS,
7  IRETMS, STROKE(10) , STRKDS(10) , STRKMS( 5) ,
8  LENGTH,CDXS,CDYS,CDZS , NTYPE , RAD(4), SS(4) , ATTHCK(3),
9  ATTPRS(3) , ADIST , SOILP(3) , NMODES,
A  COEF , GAMMA , FRIC , SCMAX , STMAX , CDC(5), CDT(5),
B  SRULT , SRULC , SRT(5), SRC(5), PFT(5), PFC(5), GFLEGS(5),
C  CURMSV, CURMSL, INDFXD, INDFYD, INDFZD, INDFXR, INDFYR,
D  INDFZR, TIMAX , DRAGST, IFP
COMMON
1CMS, CDCONT, SOILNU, SLRHO, NOOUT, XOUT(10), YOUT(10), ZOUT(10),
2MODEIN, POUTX(10,5), POUTY(10,5), POUTZ(10,5), PCGX(5), PCGY(5),
3PCGZ(5), AEI1, AEI2, FORMS(5), FORDS(10), FORCE, NFORC, SAVMSX(5),
4SAVMSZ(5), SAVDSX(10), SAVDSY(10), SAVDSZ(10), IQUOUT, GSINZT,
5GCOSZT, STAB, STABVL, ISTAB, JCKSAB, VELX, VELY, VELZ, SAVMSY(5)
COMMON
1  SMXMSC(5), TMXMSC(5), SMXMST(5), TMXMST(5),
2  SMXDSC(10), TMXDSC(10), SMXDST(10), TMXDST(10)
3  ,SLNGMS(5), SLNGDS(10), CURDSL, INLEG , IFPPRT,
4  IMPACT(5), IPRTFP(5), KOUNT(5) ,ANGX, ANGY, ANGZ
C
C  SUBROUTINE INITIALIZATION
C
IF(NMODES.EQ.0)GO TO 11
DO 10 I=1,NMODES
GFLEGS(I)=0.0
10 CONTINUE
11 CONTINUE
C
C  * * * * *
C  DETERMINE LANDING GEAR STRUT LOADS
C  * * * * *
C
C  * * * * *
C  MAIN STRUT

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C	* * * * *	FTPD1120
C		FTPD1130
	IMS=1	FTPD1140
C		FTPD1150
C	CENTER BODY STRUT ATTACH POINT IN L.C.S.	FTPD1160
C		FTPD1170
	XT=XMSCB(IFP)	FTPD1180
	YT=YMSCB(IFP)	FTPD1190
	ZT=ZMSCB(IFP)	FTPD1200
	IF(NMODES.EQ.0)GO TO 101	FTPD1210
	DO 100 I=1,NMODES	FTPD1220
	XT=XT+PMSX(IFP,I)*GC(1,I)	FTPD1230
	YT=YT+PMSY(IFP,I)*GC(1,I)	FTPD1240
100	ZT=ZT+PMSZ(IFP,I)*GC(1,I)	FTPD1250
101	CONTINUE	FTPD1260
	XTMS=XT	FTPD1270
	YTMS=YT	FTPD1280
	ZTMS=ZT	FTPD1290
C		FTPD1300
C	CENTER BODY STRUT ATTACH POINT IN S.C.S.	FTPD1310
C		FTPD1320
	PVCBX=XS+DC(1,1)*XT+DC(1,2)*YT+DC(1,3)*ZT	FTPD1330
	PVCBY=YS+DC(2,1)*XT+DC(2,2)*YT+DC(2,3)*ZT	FTPD1340
	PVCBZ=ZS+DC(3,1)*XT+DC(3,2)*YT+DC(3,3)*ZT	FTPD1350
C		FTPD1360
C	FOOTPAD END STRUT ATTACH POINT IN S.C.S.	FTPD1370
C		FTPD1380
	PVFPX = XFPS(1,IFP)	FTPD1390
	PVFPY = YFPS(1,IFP)	FTPD1400
	PVFPZ = ZFPS(1,IFP)	FTPD1410
C		FTPD1420
C	SET UP LOAD-STROKE CURVE	FTPD1430
C		FTPD1440
	DO 115 I=1,5	FTPD1450
	PFC(I)=PFCMS(I)	FTPD1460
	PFT(I)=PFTMS(I)	FTPD1470
	SRC(I)=SRCMS(I)	FTPD1480
	SRT(I)=SRTMS(I)	FTPD1490
	CDC(I)=CDCMS(I)	FTPD1500
	CDT(I)=CDTMS(I)	FTPD1510
115	CONTINUE	FTPD1520
	COEF=COEFMS	FTPD1530
	GAMMA=GAMMS	FTPD1540
	SRULC=SRUCMS	FTPD1550
	SRULT=SRUTMS	FTPD1560
	SCMAX=SCMXMS	FTPD1570
	STMAX=STMXMS	FTPD1580
	FRIC=FRICMS	FTPD1590
	SLO=SLOMS(IFP)	FTPD1600
C		FTPD1610
C	SET UP INDICATORS	FTPD1620
C		FTPD1630
	STRP= STRPMS(IFP)	FTPD1640
	SPNGC= URCMS(IFP)	FTPD1650
	SPNGT= URTMS(IFP)	FTPD1660
	DISTC= SETCMS(IFP)	FTPD1670

	DISTT= SETTMS(IFP)	FTPD1680
	PFORC= PRFCMS(IFP)	FTPD1690
	PFORT= PRFTMS(IFP)	FTPD1700
	FOREVC= FRVMSC(IFP)	FTPD1710
	FOREVT= FRVMST(IFP)	FTPD1720
	IPOSC= IPOCMS(IFP)	FTPD1730
	IPOST= IPOTMS(IFP)	FTPD1740
	INDULC= INDCMS(IFP)	FTPD1750
	INDULT= INDTMS(IFP)	FTPD1760
	IPREV= IPRMS(IFP)	FTPD1770
	IRET=IRETMS	FTPD1780
C		FTPD1790
	CALL STRUT(STRP,SPNGC,SPNGT,DISTC,DISTT,PFORC,PFORT,FOREVC,	FTPD1800
	* FOREVT,IPOSC,IPOST,INDULC,INDULT,IPREV,IRET,IFP)	FTPD1810
C		FTPD1820
C	SAVE INDICATORS	FTPD1830
C		FTPD1840
	IF(IBOTM.NE.0) RETURN	FTPD1850
	STRKMS(IFP)=STROKE(IFP)	FTPD1860
	FORMS(IFP)=FORCE	FTPD1870
	IF(TIME.EQ.0.0) GO TO 117	FTPD1880
	IF(IFIN.NE.0) GO TO 117	FTPD1890
	STRPMS(IFP)=STROKE(IFP)	FTPD1900
	SLNGMS(IFP)=CURMSL	FTPD1910
	URCMS(IFP) =SPNGC	FTPD1920
	URTMS(IFP) =SPNGT	FTPD1930
	SETCMS(IFP) =DISTC	FTPD1940
	SETTMS(IFP) =DISTT	FTPD1950
	PRFCMS(IFP) =PFORC	FTPD1960
	PRFTMS(IFP) =PFORT	FTPD1970
	FRVMSC(IFP) =FOREVC	FTPD1980
	FRVMST(IFP) =FOREVT	FTPD1990
	IPOCMS(IFP) =IPOSC	FTPD2000
	IPOTMS(IFP) =IPOST	FTPD2010
	INDCMS(IFP) =INDULC	FTPD2020
	INDTMS(IFP) =INDULT	FTPD2030
	IPRMS(IFP) =IPREV	FTPD2040
C	SAVE MAXIMUM STROKE	FTPD2050
	IF(STROKE(IFP).GT.SMXMSC(IFP))GO TO 118	FTPD2060
	SMXMSC(IFP)=STROKE(IFP)	FTPD2070
	TMXMSC(IFP)=TIME	FTPD2080
	GO TO 117	FTPD2090
118	IF(STROKE(IFP).LT.SMXMST(IFP))GO TO 117	FTPD2100
	SMXMST(IFP)=STROKE(IFP)	FTPD2110
	TMXMST(IFP)=TIME	FTPD2120
117	CONTINUE	FTPD2130
C		FTPD2140
C	SAVE FORCES AND TORQUES	FTPD2150
C		FTPD2160
	FX=-DC(1,1)*FSTX-DC(2,1)*FSTY-DC(3,1)*FSTZ	FTPD2170
	FY=-DC(1,2)*FSTX-DC(2,2)*FSTY-DC(3,2)*FSTZ	FTPD2180
	FZ=-DC(1,3)*FSTX-DC(2,3)*FSTY-DC(3,3)*FSTZ	FTPD2190
	IF(NFORC.EQ.0) GO TO 116	FTPD2200
	SAVMSX(IFP)=FX	FTPD2210
	SAVMSY(IFP)=FY	FTPD2220
	SAVMSZ(IFP)=FZ	FTPD2230

116	CONTINUE	FTPD2240
	FSXSI(IFP)=FSTX	FTPD2250
	FSYSI(IFP)=FSTY	FTPD2260
	FSZSI(IFP)=FSTZ	FTPD2270
	FLXS=-FSTX	FTPD2280
	FLYS=-FSTY	FTPD2290
	FLZS=-FSTZ	FTPD2300
	TLXL=YT*FZ-ZT*FY	FTPD2310
	TLYL=ZT*FX-XT*FZ	FTPD2320
	TLZL=XT*FY-YT*FX	FTPD2330
	IF(NMODES.EQ.0)GO TO 121	FTPD2340
	DO 120 I=1,NMODES	FTPD2350
120	GFLEGS(I)=GFLEGS(I)+FX*PMSX(IFP,I)+FY*PMSY(IFP,I)+FZ*PMSZ(IFP,I)	FTPD2360
121	CONTINUE	FTPD2370
C		FTPD2380
C	* * * * *	FTPD2390
C	DRAG STRUT	FTPD2400
C	* * * * *	FTPD2410
C		FTPD2420
	IMS=0	FTPD2430
	DO 2500 III=1,2	FTPD2440
	NNN=2*(IFP-1)+III	FTPD2450
	SRBEND=0.	FTPD2460
C		FTPD2470
C	CENTER BODY STRUT ATTACH POINT IN L.C.S.	FTPD2480
C		FTPD2490
	XT=XDSCB(NNN)	FTPD2500
	YT=YDSCB(NNN)	FTPD2510
	ZT=ZDSCB(NNN)	FTPD2520
	IF(NMODES.EQ.0)GO TO 2101	FTPD2530
	DO 2100 I=1,NMODES	FTPD2540
	XT=XT+PDSX(NNN,I)*GC(1,I)	FTPD2550
	YT=YT+PDSY(NNN,I)*GC(1,I)	FTPD2560
2100	ZT=ZT+PDSZ(NNN,I)*GC(1,I)	FTPD2570
2101	CONTINUE	FTPD2580
C		FTPD2590
C	CENTER BODY ATTACH POINT IN S.C.S.	FTPD2600
C		FTPD2610
	PVCBX=XS+DC(1,1)*XT+DC(1,2)*YT+DC(1,3)*ZT	FTPD2620
	PVCBY=YS+DC(2,1)*XT+DC(2,2)*YT+DC(2,3)*ZT	FTPD2630
	PVCBZ=ZS+DC(3,1)*XT+DC(3,2)*YT+DC(3,3)*ZT	FTPD2640
	IF(ILEG.EQ.0)GO TO 2000	FTPD2650
C		FTPD2660
C	CANTILEVER GEAR	FTPD2670
C		FTPD2680
C	FOOTPAD END STRUT ATTACH POINT IN S.C.S.	FTPD2690
C		FTPD2700
	DEL=CURMSL-DRAGST	FTPD2710
	IF(DEL.LT.0.0) DEL=0.0	FTPD2720
	PVFPX = XFPS(1,IFP)+CDXS*DEL	FTPD2730
	PVFPY = YFPS(1,IFP)+CDYS*DEL	FTPD2740
	PVFPZ = ZFPS(1,IFP)+CDZS*DEL	FTPD2750
C		FTPD2760
C	MAIN STRUT BENDING	FTPD2770
C		FTPD2780
	IF(AEI1.LE.0.0.OR.AEI2.LE.0.0) GO TO 2000	FTPD2790

DX=PVFPX-PVCBX	FTPD2800
DY=PVFPY-PVCBY	FTPD2810
DZ=PVFPZ-PVCBZ	FTPD2820
SSL=SQRT(DX*DX+DY*DY+DZ*DZ)	FTPD2830
SX=DX/SSL	FTPD2840
SY=DY/SSL	FTPD2850
SZ=DZ/SSL	FTPD2860
SDOTM=CDXS*SX+CDYS*SY+CDZS*SZ	FTPD2870
SDOTN=1.0-SDOTM**2	FTPD2950
SN2=SDOTN**2	FTPD2960
D2=DRAGST**2	FTPD2970
D3=DRAGST**3	FTPD2980
AAA=(D3*DEL)/(3*CURMSL)*(1.-DRAGST/CURMSL)	FTPD2990
BBB=D2*(D2/CURMSL-D3/(3.*CURMSL**2)+CURMSL/3.-DRAGST)	FTPD3000
SRBEND=1./(SN2*((AAA/AEI1)+(BBB/AEI2)))	FTPD3010
2000 CONTINUE	FTPD3020
C	FTPD3030
C SET UP LOAD-STROKE CURVE	FTPD3040
C	FTPD3050
DO 2115 I=1,5	FTPD3060
PFC(I)=PFCDS(I)	FTPD3070
PFT(I)=PFTDS(I)	FTPD3080
IF(ILEG.EQ.0) GO TO 2114	FTPD3090
IF(AEI1.LE.0.0.OR.AEI2.LE.0.0) GO TO 2114	FTPD3100
SRC(I)=(SRBEND*SRCDS(I))/(SRBEND+SRCDS(I))	FTPD3110
SRT(I)=(SRBEND*SRTDS(I))/(SRBEND+SRTDS(I))	FTPD3120
CDC(I)=CDCDS(I)+PFCDS(I)/SRBEND	FTPD3130
CDT(I)=CDTDS(I)+PFTDS(I)/SRBEND	FTPD3140
GO TO 2115	FTPD3150
2114 SRC(I)=SRCDS(I)	FTPD3160
SRT(I)=SRTDS(I)	FTPD3170
CDC(I)=CDCDS(I)	FTPD3180
CDT(I)=CDTDS(I)	FTPD3190
2115 CONTINUE	FTPD3200
COEF=COEFDS	FTPD3210
GAMMA=GAMDS	FTPD3220
SRULC=SRUCDS	FTPD3230
SRULT=SRUTDS	FTPD3240
SCMAX=SCMXDS	FTPD3250
STMAX=STMXDS	FTPD3260
FRIC=FRICDS	FTPD3270
SLO=SLODS(NNN)	FTPD3280
C	FTPD3290
C SET UP INDICATORS	FTPD3300
C	FTPD3310
STRP= STRPDS(NNN)	FTPD3320
SPNGC= URCDSD(NNN)	FTPD3330
SPNGT= URTDS(NNN)	FTPD3340
DISTC= SETCDS(NNN)	FTPD3350
DISTT= SETTDS(NNN)	FTPD3360
PFORC= PRFCDS(NNN)	FTPD3370
PFORT= PRFTDS(NNN)	FTPD3380
FOREVC= FRVDSC(NNN)	FTPD3390
FOREVT= FRVDST(NNN)	FTPD3400
IPOSC= IPOCDSD(NNN)	FTPD3410
IPOST= IPOTDS(NNN)	FTPD3420

	INDULC= INDCDS(NNN)	FTP3430
	INDULT= INDTDS(NNN)	FTP3440
	IPREV= IPRDS(NNN)	FTP3450
	IRET=IRETDS	FTP3460
C		FTP3470
	CALL STRUT(STRP,SPNGC,SPNGT,DISTC,DISTT,PFORC,PFORT,FOREVC,	FTP3480
	* FOREVT,IPOSC,IPOST,INDULC,INDULT,IPREV,IRET,NNN)	FTP3490
C		FTP3500
C	SAVE INDICATORS	FTP3510
C		FTP3520
	IF(IBOTM.NE.0)IFP=NNN*100	FTP3530
	IF(IBOTM.NE.0) RETURN	FTP3540
	STRKDS(NNN)=STROKE(NNN)	FTP3550
	FORDS(NNN)=FORCE	FTP3560
	IF(TIME.EQ.0.0) GO TO 2127	FTP3570
	IF(IFIN.NE.0) GO TO 2127	FTP3580
	STRPDS(NNN)=STROKE(NNN)	FTP3590
	SLNGDS(NNN)=CURDSL	FTP3600
	URCDS(NNN) =SPNGC	FTP3610
	URTDS(NNN) =SPNGT	FTP3620
	SETCDS(NNN) =DISTC	FTP3630
	SETTDS(NNN) =DISTT	FTP3640
	PRFCDS(NNN) =PFORC	FTP3650
	PRFTDS(NNN) =PFORT	FTP3660
	FRVDSC(NNN) =FOREVC	FTP3670
	FRVDST(NNN) =FOREVT	FTP3680
	IPOCDS(NNN) =IPOSC	FTP3690
	IPOTDS(NNN) =IPOST	FTP3700
	INDCDS(NNN) =INDULC	FTP3710
	INDTDS(NNN) =INDULT	FTP3720
	IPRDS(NNN) =IPREV	FTP3730
C	SAVE MAXIMUM STROKE	FTP3740
	IF(STROKE(NNN).GT.SMXDSC(NNN))GO TO 2128	FTP3750
	SMXDSC(NNN)=STROKE(NNN)	FTP3760
	TMXDSC(NNN)=TIME	FTP3770
	GO TO 2127	FTP3780
2128	IF(STROKE(NNN).LT.SMXDST(NNN))GO TO 2127	FTP3790
	SMXDST(NNN)=STROKE(NNN)	FTP3800
	TMXDST(NNN)=TIME	FTP3810
2127	CONTINUE	FTP3820
	IF(ILEG.EQ.0)GO TO 2119	FTP3830
C		FTP3840
C	SAVE FORCES AND TORQUES - CANTILEVER	FTP3850
C		FTP3860
	FX=-DC(1,1)*FSTX-DC(2,1)*FSTY-DC(3,1)*FSTZ	FTP3870
	FY=-DC(1,2)*FSTX-DC(2,2)*FSTY-DC(3,2)*FSTZ	FTP3880
	FZ=-DC(1,3)*FSTX-DC(2,3)*FSTY-DC(3,3)*FSTZ	FTP3890
	IF(NFORC.EQ.0) GO TO 2118	FTP3900
	SAVDSX(NNN)=FX	FTP3910
	SAVDSY(NNN)=FY	FTP3920
	SAVDSZ(NNN)=FZ	FTP3930
2118	CONTINUE	FTP3940
	FLXS=FLXS-FSTX	FTP3950
	FLYS=FLYS-FSTY	FTP3960
	FLZS=FLZS-FSTZ	FTP3970
	TLXL=TLXL+YT*FZ-ZT*FY	FTP3980

	TLYL=TLYL+ZT*FX-XT*FZ	FTPD3990
	TLZL=TLZL+XT*FY-YT*FX	FTPD4000
	IF(NMODES.EQ.0)GO TO 2117	FTPD4010
	DO 2116 I=1,NMODES	FTPD4020
2116	GFLEGS(I)=GFLEGS(I)+FX*PDSX(NNN,I)+FY*PDSY(NNN,I)+FZ*PDSZ(NNN,I)	FTPD4030
2117	CONTINUE	FTPD4040
	FA=FSTX*CDXS+FSTY*CDYS+FSTZ*CDZS	FTPD4050
	FAX=FA*CDXS	FTPD4060
	FAY=FA*CDYS	FTPD4070
	FAZ=FA*CDZS	FTPD4080
	FLX=FSTX-FAX	FTPD4090
	FLY=FSTY-FAY	FTPD4100
	FLZ=FSTZ-FAZ	FTPD4110
	RAT2=(CURMSL-DRAGST)/CURMSL	FTPD4120
	IF(RAT2.LT.0.0)RAT2=0.0	FTPD4130
	RAT1=1.-RAT2	FTPD4140
	FSXSI(IFP)=FSXSI(IFP)+RAT1*FLX	FTPD4150
	FSYSI(IFP)=FSYSI(IFP)+RAT1*FLY	FTPD4160
	FSZSI(IFP)=FSZSI(IFP)+RAT1*FLZ	FTPD4170
	FXX=FAX+RAT2*FLX	FTPD4180
	FYY=FAY+RAT2*FLY	FTPD4190
	FZZ=FAZ+RAT2*FLZ	FTPD4200
	FX=DC(1,1)*FXX+DC(1,2)*FYY+DC(1,3)*FZZ	FTPD4210
	FZ=DC(3,1)*FXX+DC(3,2)*FYY+DC(3,3)*FZZ	FTPD4220
	FY=DC(2,1)*FXX+DC(2,2)*FYY+DC(2,3)*FZZ	FTPD4230
	IF(NFORC.EQ.0) GO TO 2122	FTPD4240
	SAVMSX(IFP)=SAVMSX(IFP)+FX	FTPD4250
	SAVMSY(IFP)=SAVMSY(IFP)+FY	FTPD4260
	SAVMSZ(IFP)=SAVMSZ(IFP)+FZ	FTPD4270
2122	CONTINUE	FTPD4280
	FLXS=FLXS+FXX	FTPD4290
	FLYS=FLYS+FYY	FTPD4300
	FLZS=FLZS+FZZ	FTPD4310
	TLXL=TLXL+YTMS*FZ-ZTMS*FY	FTPD4320
	TLYL=TLYL+ZTMS*FX-XTMS*FZ	FTPD4330
	TLZL=TLZL+XTMS*FY-YTMS*FX	FTPD4340
	IF(NMODES.EQ.0)GO TO 2201	FTPD4350
	DO 2200 I=1,NMODES	FTPD4360
2200	GFLEGS(I)=GFLEGS(I)+FX*PMSX(IFP,I)+FY*PMSY(IFP,I)+FZ*PMSZ(IFP,I)	FTPD4370
2201	CONTINUE	FTPD4380
	GO TO 2500	FTPD4390
C		FTPD4400
C	SAVE FORCES AND TORQUES - INVERTED TRIPOD	FTPD4410
C		FTPD4420
2119	CONTINUE	FTPD4430
	FX=-DC(1,1)*FSTX-DC(2,1)*FSTY-DC(3,1)*FSTZ	FTPD4440
	FY=-DC(1,2)*FSTX-DC(2,2)*FSTY-DC(3,2)*FSTZ	FTPD4450
	FZ=-DC(1,3)*FSTX-DC(2,3)*FSTY-DC(3,3)*FSTZ	FTPD4460
	IF(NFORC.EQ.0) GO TO 2123	FTPD4470
	SAVDSY(NNN)=FY	FTPD4480
	SAVDSZ(NNN)=FZ	FTPD4490
	SAVDSX(NNN)=FX	FTPD4500
2123	CONTINUE	FTPD4510
	FSXSI(IFP)=FSXSI(IFP)+FSTX	FTPD4520
	FSYSI(IFP)=FSYSI(IFP)+FSTY	FTPD4530
	FSZSI(IFP)=FSZSI(IFP)+FSTZ	FTPD4540

FLXS=FLXS-FSTX	FTPD4550
FLYS=FLYS-FSTY	FTPD4560
FLZS=FLZS-FSTZ	FTPD4570
TLXL=TLXL+YT*FZ-ZT*FY	FTPD4580
TLYL=TLYL+ZT*FX-XT*FZ	FTPD4590
TLZL=TLZL+XT*FY-YT*FX	FTPD4600
IF(NMODES.EQ.0)GO TO 2121	FTPD4610
DO 2120 I=1,NMODES	FTPD4620
2120 GFLEGS(I)=GFLEGS(I)+FX*PDSX(NNN,I)+FY*PDSY(NNN,I)+FZ*PDSZ(NNN,I)	FTPD4630
2121 CONTINUE	FTPD4640
2500 CONTINUE	FTPD4650
C	FTPD4660
C * * * * *	FTPD4670
C DETERMINE SOIL FORCES	FTPD4680
C * * * * *	FTPD4690
3000 CONTINUE	FTPD4700
IF(XFPS(1,IFP)-ATTH(IFP).GT.0.0)GO TO 5000	FTPD4710
C	FTPD4720
C FOOTPAD IMPACT PRINT CONTROL	FTPD4730
C	FTPD4740
IF(IFPPRT.EQ.0)GO TO 4010	FTPD4750
IF(IFIN.NE.0)GO TO 4010	FTPD4760
IF(IPTCNT.EQ.1)GO TO 4010	FTPD4770
IF(IMPACT(IFP).NE.0)GO TO 4000	FTPD4780
IMPACT(IFP)=1	FTPD4790
IPRTFP(IFP)=1	FTPD4800
4000 CONTINUE	FTPD4810
IF(IMPACT(IFP).EQ.2)GO TO 4010	FTPD4820
KOUNT(IFP)=KOUNT(IFP)+1	FTPD4830
IF(KOUNT(IFP).LE.IFPPRT)GO TO 4010	FTPD4840
IPRTFP(IFP)=IPTCNT	FTPD4850
KOUNT(IFP)=0	FTPD4860
IMPACT(IFP)=2	FTPD4870
4010 CONTINUE	FTPD4880
C	FTPD4890
C CALL SOIL	FTPD4900
C	FTPD4910
C CHECK MAGNITUDE OF IN PLANE SOIL FORCE	FTPD4920
C	FTPD4930
FF=SQRT(SOILY(IFP)**2+SOILZ(IFP)**2)	FTPD4940
IF(FF.LT.0.00001) GO TO 3200	FTPD4950
CFFY=SOILY(IFP)/FF	FTPD4960
CFFZ=SOILZ(IFP)/FF	FTPD4970
FH=SQRT(FSYSI(IFP)**2+FSZSI(IFP)**2)	FTPD4980
IF(FH.GT.0.00001) GO TO 3102	FTPD4990
HDOTF=0.0	FTPD5000
GO TO 3103	FTPD5010
3102 CFHY=FSYSI(IFP)/FH	FTPD5020
CFHZ=FSZSI(IFP)/FH	FTPD5030
HDOTF=CFHY*CFFY+CFHZ*CFFZ	FTPD5040
3103 VS=SQRT(YFPSD(1,IFP)**2+ZFPSD(1,IFP)**2)	FTPD5050
AMV=FPMASS*VS/CURDT	FTPD5060
IF(FF.LE.(-FH*HDOTF+AMV)) GO TO 3200	FTPD5070
IF(HDOTF.LE.0.)FF=-FH*HDOTF+AMV	FTPD5080
IF(HDOTF.GT.0.)FF=AMV	FTPD5090
IF(FF.LT.0.)FF=0.0	FTPD5100

SOILY(IFP)=CFFY*FF	FTPD5110
SOILZ(IFP) =CFFZ*FF	FTPD5120
3200 CONTINUE	FTPD5130
RETURN	FTPD5140
C	FTPD5150
C * * * * *	FTPD5160
C FOOTPAD OFF GROUND	FTPD5170
C * * * * *	FTPD5180
C	FTPD5190
5000 CONTINUE	FTPD5200
SOILX(IFP)=0.0	FTPD5210
SOILY(IFP)=0.0	FTPD5220
SOILZ(IFP)=0.0	FTPD5230
C	FTPD5240
C FOOTPAD IMPACT PRINT CONTROL	FTPD5250
C	FTPD5260
IF(IFPPRT.EQ.0)GO TO 5020	FTPD5270
IF(IFIN.NE.0)GO TO 5020	FTPD5280
IF(IMPACT(IFP).NE.1)GO TO 5010	FTPD5290
KOUNT(IFP)=KOUNT(IFP)+1	FTPD5300
IF(KOUNT(IFP).LE.IFPPRT)GO TO 5020	FTPD5310
IPRTFP(IFP)=IPTCNT	FTPD5320
KOUNT(IFP)=0	FTPD5330
5010 IMPACT(IFP)=0	FTPD5340
5020 CONTINUE	FTPD5350
RETURN	FTPD5360
END	FTPD5370

	SUBROUTINE GEOM	GEOM 10
C	THIS SUBROUTINE DETERMINES THE DIRECTION COSINE MATRIX AND	GEOM 20
C	DETERMINES THE TIME DERIVATES OF THE EULER ANGLES	GEOM 30
	COMMON COMINT(400)	GEOM 40
	EQUIVALENCE (COMINT(36), PHI)	GEOM 50
	EQUIVALENCE (COMINT(40), PHID)	GEOM 60
	EQUIVALENCE (COMINT(44), WX)	GEOM 70
	EQUIVALENCE (COMINT(52), THTA)	GEOM 80
	EQUIVALENCE (COMINT(56), THTAD)	GEOM 90
	EQUIVALENCE (COMINT(60), WY)	GEOM 100
	EQUIVALENCE (COMINT(68), PSI)	GEOM 110
	EQUIVALENCE (COMINT(72), PSID)	GEOM 120
	EQUIVALENCE (COMINT(76), WZ)	GEOM 130
	COMMON	GEOM 140
1	CBMASS, CBIXX , CBIXZ , CBIYY , CBIYZ , CBIZZ , FPMASS,CBIXY,	GEOM 150
2	DC(3,3)	GEOM 160
	COSPHI = COS(PHI)	GEOM 170
	SINPHI = SIN(PHI)	GEOM 180
	COSTHA = COS(THTA)	GEOM 190
	SINTHA = SIN(THTA)	GEOM 200
	COSPSI = COS(PSI)	GEOM 210
	SINPSI = SIN(PSI)	GEOM 220
	DC(1,1) = COSTHA * COSPSI	GEOM 230
	A = SINTHA * COSPSI	GEOM 240
	B = COSPHI * SINPSI	GEOM 250
	DC(1,2) = SINPHI * A - B	GEOM 260
	C = SINPHI * SINPSI	GEOM 270
	DC(1,3) = COSPHI * A + C	GEOM 280
	DC(2,1) = COSTHA * SINPSI	GEOM 290
	DC(2,2) = SINTHA * C + COSPHI*COSPSI	GEOM 300
	DC(2,3) = SINTHA * B - SINPHI*COSPSI	GEOM 310
	DC(3,1) = - SINTHA	GEOM 320
	DC(3,2) = SINPHI*COSTHA	GEOM 330
	DC(3,3) = COSPHI*COSTHA	GEOM 340
	*****	GEOM 350
C	IF(ABS(COSTHA)-0.1E-10) 100,100,200	GEOM 360
200	CONTINUE	GEOM 370
	PSID =(WZ*COSPHI + WY*SINPHI) / COSTHA	GEOM 380
	PHID = WX + SINTHA* PSID	GEOM 390
	THTAD = WY*COSPHI - WZ*SINPHI	GEOM 400
	RETURN	GEOM 410
100	CONTINUE	GEOM 420
	PSID = 0.0	GEOM 430
	PHID = WX	GEOM 440
	THTAD = SQRT(WY*WY+WZ*WZ)	GEOM 450
	RETURN	GEOM 460
	END	GEOM 470

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SUBROUTINE STRUT(STRP,SPNGC,SPNGT,DISTC,DISTT,PFORC,PFORT,FOREVC, STRU 10
1          FOREVT,IPOSC,IPOST,INDULC,INDULT,IPREV, STRU 20
2          IRET,NNN ) STRU 30
C STRU 40
C THIS SUBROUTINE DETERMINES THE FORCES IN A TYPICAL STRU 50
C LANDING GEAR STRUT STRU 60
C STRU 70
C THE STRUT FORCES ACTING ON THE FOOTPAD IN THE SURFACE STRU 80
C COORDINATE SYSTEM ARE RETURNED STRU 90
C STRU 100
DIMENSION GC(4,5), GCD(4,5), GCDD(4,5) STRU 110
DIMENSION XFPS(4,5), YFPS(4,5), ZFPS(4,5), STRU 120
1          XFPSD(4,5), YFPSD(4,5), ZFPSD(4,5), STRU 130
2          XFPSDD(4,5), YFPSDD(4,5), ZFPSDD(4,5) STRU 140
COMMON / THISV/ TIMHSA(1), ATTH(5),AM(6,6),AME1(3),INDFPI(6), STRU 150
1          INDFPC(6), STRU 160
2          STRPDS(10), STRPMS( 5), IPOCD(10), IPOCMS( 5), URCDS (10), STRU 170
3          URTDS (10), URCMS ( 5), URTMS ( 5), SETCDS(10), SETTDS(10), STRU 180
4          SETCMS( 5), SETTMS( 5), INDCDS(10), INDTDS(10), INDCMS(10), STRU 190
5          INDTMS( 5), PRFCDS (10), PRFTDS (10), PRFCMS ( 5), PRFTMS ( 5), STRU 200
6          IPRDS (10), IPRMS ( 5), FRVDSC(10), FRVDST(10), FRVMSC( 5), STRU 210
L          FRVMST( 5), IPOTDS(10), IPOTMS( 5) STRU 220
COMMON COMINT(400) STRU 230
EQUIVALENCE ( COMINT( 3 ), IBOTM ) STRU 240
EQUIVALENCE ( COMINT( 84 ), GC ) STRU 250
EQUIVALENCE ( COMINT(104 ), GCD ) STRU 260
EQUIVALENCE ( COMINT(124 ), GCDD ) STRU 270
EQUIVALENCE ( COMINT(144 ), XFPS ) STRU 280
EQUIVALENCE ( COMINT(164 ), XFPSD ) STRU 290
EQUIVALENCE ( COMINT(184 ), XFPSDD ) STRU 300
EQUIVALENCE ( COMINT(204 ), YFPS ) STRU 310
EQUIVALENCE ( COMINT(224 ), YFPSD ) STRU 320
EQUIVALENCE ( COMINT(244 ), YFPSDD ) STRU 330
EQUIVALENCE ( COMINT(264 ), ZFPS ) STRU 340
EQUIVALENCE ( COMINT(284 ), ZFPSD ) STRU 350
EQUIVALENCE ( COMINT(304 ), ZFPSDD ) STRU 360
EQUIVALENCE ( COMINT(368 ), CURDT ) STRU 370
COMMON STRU 380
1          CBMASS, CBIXX , CBIXZ , CBIYY , CBIYZ , CBIZZ , FPMASS,CBIXY, STRU 390
2          DC(3,3) , XFP(5), YFP(5), ZFP(5), WNX(5), WNY(5), STRU 400
3          WNZ(5), PX(5) , PY(5) , PZ(5) , GM(5) , OMEGA(5) , STRU 410
4          GRAV , GRAVE , ZETA , FTS (6) , FSXSI(5) , STRU 420
5          FSYSI(5) , FSZSI(5) , SOILX(5) , STRU 430
6          SOILY(5) , SOILZ(5) , PMSX(5,5) , STRU 440
7          PMSY(5,5) , PMSZ(5,5) , PDSX(10,5) , STRU 450
8          PDSY(10,5) , PDSZ(10,5) , FLXS , FLYS , FLZS , STRU 460
9          TLXL , TLYL , TLZL , SLO , XMSCB(5) , STRU 470
C          YMSCB(5) , ZMSCB(5) , XDSCB(10) , STRU 480
D          YDSCB(10) , ZDSCB(10) , ILEG , IMS , STRU 490
E          FSTX , FSTY , FSTZ , PVCBX , PVCBY , PVCBZ , STRU 500
F          PVFPX , PVFPY , PVFPZ , NOLEG , SLOMS(5) , STRU 510
G          SLODS(10) STRU 520
COMMON STRU 530
1          PFCMS(5) , PFCDS(5) , PFTDS(5) , STRU 540
2          PFTMS(5) , SRCDS(5) , SRCMS(5) , STRU 550

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3   SRTDS(5)      ,      SRTMS(5)      , COEFDS, COEFMS, GAMDS ,      STRU 560
4   GAMMS , SRUCDS, SRUCMS, SRUTDS, SRUTMS, SCMXDS, SCMXMS,      STRU 570
5   STMXDS, STMXMS, CDCDS(5)      , CDCMS(5)      ,      STRU 580
6   CDTDS(5)      , CDTMS(5)      , FRICDS, FRICMS, IRETDS,      STRU 590
7   IRETMS, STROKE(10)      , STRKDS(10)      , STRKMS( 5)      ,      STRU 600
8   LENGTH,CDXS,CDYS,CDZS , NTYPE , RAD(4) , SS(4) , ATTHCK(3) ,      STRU 610
9   ATTPRS(3)      , ADIST , SOILP(3)      , NMODES,      STRU 620
A   COEF , GAMMA , FRIC , SCMAX , STMAX , CDC(5), CDT(5),      STRU 630
B   SRULT , SRULC , SRT(5), SRC(5), PFT(5), PFC(5), GFLEGS(5),      STRU 640
C   CURMSV, CURMSL, INDFXD, INDFYD, INDFZD, INDFXR, INDFYR,      STRU 650
D   INDFZR, TIMAX , DRAGST, IFP      STRU 660
COMMON      STRU 670
1CMS, CDCONT, SOILNU, SLRHO,      NOOUT, XOUT(10), YOUT(10), ZOUT(10),      STRU 680
2MODEIN, POUTX(10,5), POUTY(10,5), POUTZ(10,5), PCGX(5), PCGY(5),      STRU 690
3PCGZ(5), AEI1, AEI2, FORMS(5), FORDS(10), FORCE, NFORC, SAVMSX(5),      STRU 700
4SAVMSZ(5), SAVDSX(10), SAVDSY(10), SAVDSZ(10), IQUOUT, GSINZT,      STRU 710
5GCOSZT, STAB, STABVL, ISTAB, JCKSAB, VELX, VELY, VELZ, SAVMSY(5)      STRU 720
COMMON      STRU 730
1   SMXMSC(5), TMXMSC(5), SMXMST(5), TMXMST(5),      STRU 740
2   SMXDSC(10), TMXDSC(10), SMXDST(10), TMXDST(10)      STRU 750
3   ,SLNGMS(5) ,SLNGDS(10), CURDSL, INLEG , IFPRT,      STRU 760
4   IMPACT(5) ,IPRTFP(5), KOUNT(5) ,ANGX, ANGY, ANGZ      STRU 770
C      STRU 780
C      INITIALIZE SUBROUTINE      STRU 790
C      STRU 800
C      DX=PVCBX-PVFPX      STRU 810
C      DY=PVCBY-PVFPY      STRU 820
C      DZ=PVCBZ-PVFPZ      STRU 830
C      SLNGTH=SQRT(DX*DX+DY*DY+DZ*DZ)      STRU 840
C      CDX=DX/SLNGTH      STRU 850
C      CDY=DY/SLNGTH      STRU 860
C      CDZ=DZ/SLNGTH      STRU 870
C      STR=SLNGTH-SLO      STRU 880
C      ABSTR=ABS(STR)      STRU 890
C      VELST=(STR-STRP)/CURDT      STRU 900
C      ABVEL=ABS(VELST)      STRU 910
C      IF(IMS.NE.1)GO TO 10      STRU 920
C      CDXS=CDX      STRU 930
C      CDYS=CDY      STRU 940
C      CDZS=CDZ      STRU 950
C      CURMSV=VELST      STRU 960
C      CURMSL=SLNGTH      STRU 970
10 CONTINUE      STRU 980
C      IF(IMS.EQ.0)CURDSL=SLNGTH      STRU 990
C      IJKC=IPOSC      STRU1000
C      IJKT=IPOST      STRU1010
C      STRU1020
C      DETERMINE STROKING DIRECTION      STRU1030
C      STRU1040
C      IF(STR.LT.0.0)GO TO 500      STRU1050
C      IF(STR.GT.0.0)GO TO 400      STRU1060
C      STRU1070
C      * * * * *      STRU1080
C      ZERO STRUT STROKE      STRU1090
C      * * * * *      STRU1100
C      STRU1110

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	FORCE=0.0	STRU1120
	GO TO 2002	STRU1130
C		STRU1140
C	* * * * *	STRU1150
C	STRUT IN TENSION	STRU1160
C	* * * * *	STRU1170
C		STRU1180
400	CONTINUE	STRU1190
	IF(ABSTR.GT.STMAX)GO TO 5001	STRU1200
C		STRU1210
C	WAS PREVIOUS STEP ON COMPRESSION SIDE	STRU1220
C		STRU1230
	IF(IPREV.EQ.0)GO TO 402	STRU1240
	IF(IPREV.NE.-1)GO TO 402	STRU1250
	IF(PFORC.EQ.0.0)GO TO 402	STRU1260
	IPREV=+1	STRU1270
	IF(IRET.EQ.1) GO TO 651	STRU1280
	GO TO 626	STRU1290
402	CONTINUE	STRU1300
	IPREV=+1	STRU1310
	IF(ABSTR.GT.DISTT)GO TO 401	STRU1320
	FORCE=0.0	STRU1330
	GO TO 2001	STRU1340
401	CONTINUE	STRU1350
	IF(STR.LT.STRP)GO TO 425	STRU1360
C		STRU1370
C	STRUT LOADING	STRU1380
C		STRU1390
	IF(SPNGT.EQ.0.0)GO TO 405	STRU1400
	FORCE=SPNGT*(ABSTR-DISTT)	STRU1410
	IF(FORCE.LT.FOREVT)GO TO 2001	STRU1420
	FOREVT=0.0	STRU1430
	INDULT=0	STRU1440
	SPNGT=0.0	STRU1450
	IF(IJKT.EQ.1)GO TO 407	STRU1460
	DISTT=CDT(IJKT-1)-PFT(IJKT-1)/SRT(IJKT)	STRU1470
	GO TO 408	STRU1480
407	DISTT=0.0	STRU1490
408	CONTINUE	STRU1500
	IF (FORCE.LT.PFT(IJKT))GO TO 405	STRU1510
	FORCE=PFT(IJKT)	STRU1520
	IF(ABSTR.LT.CDT(IJKT))GO TO 2001	STRU1530
405	CONTINUE	STRU1540
406	IF(ABSTR.LT.CDT(IJKT))GO TO 410	STRU1550
	DISTT=CDT(IJKT)-PFT(IJKT)/SRT(IJKT+1)	STRU1560
	IPOST=IPOST+1	STRU1570
	IJKT=IPOST	STRU1580
	GO TO 406	STRU1590
410	CONTINUE	STRU1600
	FORCE=SRT(IJKT)*(ABSTR-DISTT)	STRU1610
	IF(FORCE.GT.PFT(IJKT))FORCE=PFT(IJKT)	STRU1620
	GO TO 2001	STRU1630
C		STRU1640
C	STRUT UNLOADING	STRU1650
C		STRU1660
425	CONTINUE	STRU1670

	IF(IRET.EQ.1)GO TO 450	STRU1680
	IF(INDULT.NE.0)GO TO 475	STRU1690
426	SPNGT=SRULT	STRU1700
	IF(PFORT.NE.PFT(IPOST))SPNGT=SRT(IPOST)	STRU1705
	FOREVT=PFORT	STRU1710
	GO TO 470	STRU1720
450	CONTINUE	STRU1730
	IF(INDULT.NE.0)GO TO 475	STRU1740
451	CONTINUE	STRU1750
	FOREVT=PFORT	STRU1760
	IF(PFORT.EQ.PFT(IJKT))GO TO 465	STRU1770
	SPNGT=SRT(IJKT)	STRU1780
	GO TO 470	STRU1790
465	CONTINUE	STRU1800
	SPNGT=SRULT	STRU1810
	IF((IJKT+1).GT.5)GO TO 470	STRU1820
	SPNGT=SRT(IJKT+1)	STRU1830
470	CONTINUE	STRU1840
	INDULT=1	STRU1850
	DISTT=ABS(STRP)-PFORT/SPNGT	STRU1860
	IF(IPREV.EQ.-1)GO TO 602	STRU1870
475	CONTINUE	STRU1880
	FORCE=SPNGT*(ABSTR-DISTT)	STRU1890
	IF(ABSTR.LT.DISTT)FORCE=0.0	STRU1900
	GO TO 2001	STRU1910
C		STRU1920
C	* * * * *	STRU1930
C	STRUT IN COMPRESSTION	STRU1940
C	* * * * *	STRU1950
C		STRU1960
500	CONTINUE	STRU1970
	IF(ABSTR.GT.SCMAX)GO TO 5000	STRU1980
C		STRU1990
C	WAS PREVIOUS STEP ON TENSION SIDE	STRU2000
C		STRU2010
	IF(IPREV.EQ.0)GO TO 602	STRU2020
	IF(IPREV.NE.+1)GO TO 602	STRU2030
	IF(PFORT.EQ.0.0)GO TO 602	STRU2040
	IPREV=-1	STRU2050
	IF(IRET.EQ.+1)GO TO 451	STRU2060
	GO TO 426	STRU2070
602	CONTINUE	STRU2080
	IPREV=-1	STRU2090
	IF(ABSTR.GT.DISTC)GO TO 600	STRU2100
	FORCE=0.0	STRU2110
	GO TO 2000	STRU2120
600	CONTINUE	STRU2130
	IF(STR.GT.STRP)GO TO 625	STRU2140
C		STRU2150
C	STRUT LOADING	STRU2160
C		STRU2170
	IF(SPNGC.EQ.0.0)GO TO 605	STRU2180
	FORCE=SPNGC*(ABSTR-DISTC)	STRU2190
	IF(FORCE.LT.FOREVC)GO TO 2000	STRU2200
	FOREVC=0.0	STRU2210
	INDULC=0	STRU2220

	SPNGC=0.0	STRU2230
	IF(IJKC.EQ.1)GO TO 607	STRU2240
	DISTC=CDC(IJKC-1)-PFC(IJKC-1)/SRC(IJKC)	STRU2250
	GO TO 608	STRU2260
607	DISTC=0.0	STRU2270
608	CONTINUE	STRU2280
	IF(FORCE.LT.PFC(IJKC))GO TO 605	STRU2290
	FORCE=PFC(IJKC)	STRU2300
	IF(ABSTR.LT.CDC(IJKC))GO TO 2000	STRU2310
605	CONTINUE	STRU2320
606	IF(ABSTR.LT.CDC(IJKC))GO TO 610	STRU2330
	DISTC=CDC(IJKC)-PFC(IJKC)/SRC(IJKC+1)	STRU2340
	IPOSC=IPOSC+1	STRU2350
	IJKC=IPOSC	STRU2360
	GO TO 606	STRU2370
610	CONTINUE	STRU2380
	FORCE=SRC(IJKC)*(ABSTR-DISTC)	STRU2390
	IF(FORCE.GT.PFC(IJKC))FORCE=PFC(IJKC)	STRU2400
	GO TO 2000	STRU2410
C		STRU2420
C	STRUT UNLOADING	STRU2430
C		STRU2440
625	CONTINUE	STRU2450
	IF(IRET.EQ.1)GO TO 650	STRU2460
	IF(INDULC.NE.0)GO TO 675	STRU2470
626	SPNGC=SRULC	STRU2480
	IF(PFORC.NE.PFC(IPOSC))SPNGC=SRC(IPOSC)	STRU2485
	FOREVC=PFORC	STRU2490
	GO TO 670	STRU2500
650	CONTINUE	STRU2510
	IF(INDULC.NE.0)GO TO 675	STRU2520
651	CONTINUE	STRU2530
	FOREVC=PFORC	STRU2540
	IF(PFORC.EQ.PFC(IJKC))GO TO 665	STRU2550
	SPNGC=SRC(IJKC)	STRU2560
	GO TO 670	STRU2570
665	CONTINUE	STRU2580
	SPNGC=SRULC	STRU2590
	IF((IJKC+1).GT.5) GO TO 670	STRU2600
	SPNGC=SRC(IJKC+1)	STRU2610
670	CONTINUE	STRU2620
	INDULC=1	STRU2630
	DISTC=ABS(STRP)-PFORC/SPNGC	STRU2640
	IF(IPREV.EQ.+1)GO TO 402	STRU2650
675	CONTINUE	STRU2660
	FORCE=SPNGC*(ABSTR-DISTC)	STRU2670
	IF(ABSTR.LT.DISTC)FORCE=0.0	STRU2680
C		STRU2690
C	UPDATE INDICATORS	STRU2700
C		STRU2710
2000	CONTINUE	STRU2720
	PFORC=FORCE	STRU2730
	GO TO 2005	STRU2740
2001	CONTINUE	STRU2750
	PFORT=FORCE	STRU2760
	GO TO 2005	STRU2770

2002	CONTINUE	STRU2780
	PFORC=0.0	STRU2790
	PFORT=0.0	STRU2800
2005	CONTINUE	STRU2810
	STROKE(NNN)=STR	STRU2820
C		STRU2830
C	DETERMINE FRICTION AND DAMPING FORCES	STRU2840
C		STRU2850
	FRIFOR= SIGN(1.,VELST)*(FRIC+COEF*(ABVEL**GAMMA))	STRU2860
C		STRU2870
C	TOTAL STRUT FORCE	STRU2880
	FORCE=SIGN(1.,STR)*FORCE+FRIFOR	STRU2890
	FSTX=CDX*FORCE	STRU2900
	FSTY=CDY*FORCE	STRU2910
	FSTZ=CDZ*FORCE	STRU2920
	RETURN	STRU2930
C		STRU2940
C	STRUT BOTTOMED OUT ON COMPRESSION SIDE	STRU2950
C		STRU2960
5000	CONTINUE	STRU2970
	IBOTM=-1	STRU2980
	RETURN	STRU2990
C		STRU3000
C	STRUT BOTTOMED OUT ON TENSION SIDE	STRU3010
C		STRU3020
5001	CONTINUE	STRU3030
	IBOTM=+1	STRU3040
	RETURN	STRU3050
	END	STRU3060

	SUBROUTINE SOIL	SOIL 10
C		SOIL 20
C	THIS SUBROUTINE DETERMINES THE SOIL FORCES ACTING ON	SOIL 30
C	A FOOTPAD	SOIL 40
C		SOIL 50
C	THE EVALUATION OF THE FOOTPAD ATTENUATION SYSTEM FORCES	SOIL 60
C	IS ALSO INCLUDED HERE	SOIL 70
C		SOIL 80
C	THE SOIL/ATTENUATION FORCES ACTING ON THE FOOTPAD IN THE	SOIL 90
C	SURFACE COORDINATE SYSTEM ARE RETURNED	SOIL 100
C		SOIL 110
	DIMENSION GC(4,5), GCD(4,5), GCDD(4,5)	SOIL 120
	DIMENSION XFPS(4,5), YFPS(4,5), ZFPS(4,5),	SOIL 130
1	XFPSDD(4,5), YFPSDD(4,5), ZFPSDD(4,5),	SOIL 140
2	XFPSDD(4,5), YFPSDD(4,5), ZFPSDD(4,5)	SOIL 150
C		SOIL 160
	COMMON / THISV/ TIMHSA(1), ATTH(5),AM(6,6),AME1(3),INDFPI(6),	SOIL 170
1	INDFPC(6),	SOIL 180
2	STRPDS(10), STRPMS(5), IPOCD(10), IPOCMS(5), URCDS (10),	SOIL 190
3	URTDS (10), URCMS (5), URTMS (5), SETCDS(10), SETTDS(10),	SOIL 200
4	SETCMS(5), SETTMS(5), INDCDS(10), INDTDS(10), INDCMS(10),	SOIL 210
5	INDTMS(5), PRFCDS (10), PRFTDS (10), PRFCMS (5), PRFTMS (5),	SOIL 220
6	IPRDS (10), IPRMS (5), FRVDSC(10), FRVDST(10), FRVMSC(5),	SOIL 230
L	FRVMST(5), IPOTDS(10), IPOTMS(5)	SOIL 240
	COMMON COMINT(400)	SOIL 250
	EQUIVALENCE (COMINT(84), GC)	SOIL 260
	EQUIVALENCE (COMINT(104), GCD)	SOIL 270
	EQUIVALENCE (COMINT(124), GCDD)	SOIL 280
	EQUIVALENCE (COMINT(144), XFPS)	SOIL 290
	EQUIVALENCE (COMINT(164), XFPSD)	SOIL 300
	EQUIVALENCE (COMINT(184), XFPSDD)	SOIL 310
	EQUIVALENCE (COMINT(204), YFPS)	SOIL 320
	EQUIVALENCE (COMINT(224), YFPSD)	SOIL 330
	EQUIVALENCE (COMINT(244), YFPSDD)	SOIL 340
	EQUIVALENCE (COMINT(264), ZFPS)	SOIL 350
	EQUIVALENCE (COMINT(284), ZFPSD)	SOIL 360
	EQUIVALENCE (COMINT(304), ZFPSDD)	SOIL 370
	EQUIVALENCE (COMINT(324), TIME)	SOIL 380
	EQUIVALENCE (COMINT(343), IFIN)	SOIL 390
	EQUIVALENCE (COMINT(368), CURDT)	SOIL 400
	COMMON	SOIL 410
1	CBMASS, CBIXX , CBIXZ , CBIYY , CBIYZ , CBIZZ , FPMASS,CBIXY,	SOIL 420
2	DC(3,3) , XFP(5), YFP(5), ZFP(5), WNX(5), WNY(5),	SOIL 430
3	WNZ(5), PX(5), PY(5), PZ(5), GM(5), OMEGA(5) ,	SOIL 440
4	GRAV , GRAVE , ZETA , FTS (6) , FSXSI(5) ,	SOIL 450
5	FSYSI(5) , FSZSI(5) , SOILX(5) ,	SOIL 460
6	SOILY(5) , SOILZ(5) , PMSX(5,5) ,	SOIL 470
7	PMSY(5,5) , PMSZ(5,5) , PDSX(10,5) ,	SOIL 480
8	PDSY(10,5) , PDSZ(10,5) , FLXS , FLYS , FLZS ,	SOIL 490
9	TLXL , TLYL , TLZL , SLO , XMSCB(5) ,	SOIL 500
C	YMSCB(5) , ZMSCB(5) , XDSCB(10) ,	SOIL 510
D	YDSCB(10) , ZDSCB(10) , ILEG , IMS ,	SOIL 520
E	FSTX , FSTY , FSTZ , PVCBX , PVCBY , PVCBZ ,	SOIL 530
F	PVFPX , PVFPY , PVFPZ , NOLEF , SLOMS(5) ,	SOIL 540
G	SLODS(10)	SOIL 550

	COMMON	SOIL 560
1	PFCMS(5) , PFCDS(5) , PFTDS(5) ,	SOIL 570
2	PFTMS(5) , SRCDS(5) , SRCMS(5) ,	SOIL 580
3	SRTDS(5) , SRTMS(5) , COEFDS, COEFMS, GAMDS ,	SOIL 590
4	GAMMS , SRUCDS, SRUCMS, SRUTDS, SRUTMS, SCM XDS, SCM XMS,	SOIL 600
5	STM XDS, STM XMS, CDCDS(5) , CDCMS(5) ,	SOIL 610
6	CDTDS(5) , CDTMS(5) , FRICDS, FRICMS, IRETDS,	SOIL 620
7	IRETMS, STROKE(10) , STRKDS(10) , STRKMS(5) ,	SOIL 630
8	LENGTH,CDXS,CDYS,CDZS , NTYPE, RAD(4), SS(4) , ATTHCK(3),	SOIL 640
9	ATTPRS(3) , ADIST , SOILP(3) , NMODES,	SOIL 650
A	COEF , GAMMA , FRIC , SCMAX , STMAX , CDC(5), CDT(5),	SOIL 660
B	SRULT , SRULC , SRT(5), SRC(5), PFT(5), PFC(5), GFLEGS(5),	SOIL 670
C	CURMSV, CURMSL, INDFXD, INDFYD, INDFZD, INDFXR, INDFYR,	SOIL 680
D	INDFZR, TIMAX , DRAGST, IFP	SOIL 690
	COMMON	SOIL 700
	1CMS, CDCONT, SOILNU, SLRHO, NOOUT, XOUT(10), YOUT(10), ZOUT(10),	SOIL 710
	2MODEIN, POUTX(10,5), POUTY(10,5), POUTZ(10,5), PCGX(5), PCGY(5),	SOIL 720
	3PCGZ(5), AEI1, AEI2, FORMS(5), FORDS(10), FORCE, NFORC, SAVMSX(5),	SOIL 730
	4SAVMSZ(5), SAVDSX(10), SAVDSY(10), SAVDSZ(10), IQUOUT, GSINZT,	SOIL 740
	5GCOSZT, STAB, STABVL, ISTAB, JCKSAB, VELX, VELY, VELZ, SAVMSY(5)	SOIL 750
	COMMON	SOIL 760
1	SMXMSC(5), TMXMSC(5), SMXMST(5), TMXMST(5),	SOIL 770
2	SMXDSC(10), TMXDSC(10), SMXDST(10), TMXDST(10)	SOIL 780
3	,SLNGMS(5), SLNGDS(10), CURDSL, INLEG, IFPRT,	SOIL 790
4	IMPACT(5), IPRTFP(5), KOUNT(5) ,ANGX, ANGY, ANGZ	SOIL 800
C		SOIL 810
C	SUBROUTINE INITIALIZATION	SOIL 820
C		SOIL 830
	KJI=1	SOIL 840
	IJK=1	SOIL 850
	PIE=3.14159265	SOIL 860
	PIE2=PIE/2.	SOIL 870
	DEPTH=XFPS(1,IFP)-ATTH(IFP)	SOIL 880
	ADEPTH=ABS(DEPTH)	SOIL 890
	FPSTR=ATTH(IFP)-ADEPTH	SOIL 900
	IF(DEPTH.GE.0.0)GO TO 2000	SOIL 910
C		SOIL 920
C	FOOTPRINT AREA	SOIL 930
C		SOIL 940
	IF(FPSTR.LE.SS(4)) GO TO 101	SOIL 950
100	IJK=IJK+1	SOIL 960
	IF(IJK.GT.4) GO TO 101	SOIL 970
	IF(FPSTR.LT.SS(IJK)) GO TO 100	SOIL 980
	IF(FPSTR.EQ.SS(IJK))GO TO 102	SOIL 990
	DRDD=(RAD(IJK)-RAD(IJK-1))/(SS(IJK-1)-SS(IJK))	SOIL1000
	RR=RAD(IJK)-DRDD*(FPSTR-SS(IJK))	SOIL1010
	GO TO 104	SOIL1020
101	RR=RAD(4)	SOIL1030
	GO TO 103	SOIL1040
102	RR=RAD(IJK)	SOIL1050
103	DRDD=0.0	SOIL1060
104	AREA=PIE*RR*RR	SOIL1070
	IF(NTYPE.EQ.0) GO TO 500	SOIL1080
C		SOIL1090
C	* * * * *	SOIL1100
C	SECONDARY SOIL MECHANICS	SOIL1110

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C      * * * * *
C
PRESS=ADEPTH*SOILP(2)
IF(PRESS.GT.SOILP(3)) PRESS=SOILP(3)
SOILX(IFP)=PRESS*AREA
C
C      CHECK MAGNITUDE OF SOIL FORCE
C
SOILCR=FPMASS*(GCOSZT-XFPSD(1,IFP)/CURDT)-FSXSI(IFP)
IF(SOILCR.LT.0.0)SOILCR=0.0
IF(SOILX(IFP).GT.SOILCR)SOILX(IFP)=SOILCR
PRESS=SOILX(IFP)/AREA
150 CONTINUE
VBAR=SQRT(YFPSD(1,IFP)**2+ZFPSD(1,IFP)**2)
IF(VBAR.LT.0.000001)GO TO 200
COEF=SOILP(1)
CY=COEF*YFPSD(1,IFP)/VBAR
CZ=COEF*ZFPSD(1,IFP)/VBAR
GO TO 210
200 CY=0.0
CZ=0.0
210 CONTINUE
SOILY(IFP)=-CY*SOILX(IFP)
SOILZ(IFP)=-CZ*SOILX(IFP)
GO TO 1000
C
C      * * * * *
C      PRIMARY SOIL MECHANICS
C      * * * * *
C
500 CONTINUE
VBAR=SQRT(XFPSD(1,IFP)**2+YFPSD(1,IFP)**2+ZFPSD(1,IFP)**2)
VHOR=SQRT(YFPSD(1,IFP)**2+ZFPSD(1,IFP)**2)
THETAL=0.
IF(VBAR.GT.0.000001)THETAL=ASIN(VHOR/VBAR)
IF(XFPSD(1,IFP).GT.0.)THETAL=PIE-THETAL
COST=ABS(COS(THETAL))
SINT=SIN(THETAL)
ATHICK=RR*ADEPTH
ATHTA=AREA*COST+ATHICK*SINT
APERP=AREA*SINT+ATHICK*COST
FPHI=1.-2.*THETAL/PIE
ALAM =.25*(APERP/ATHTA)*(1.-EXP(-50.*THETAL))*(1.+SINT)
IF(THETAL.LT.PIE2) GO TO 505
FPHI=0.
ALAM =.5*(APERP/ATHTA)
505 CONTINUE
IF(THETAL.GT.(3.*PIE/4.))ALAM =0.0
CD=.8+CDCONT*RR*RR*FPHI
AAA=CD*SLRHO*ATHTA*VBAR*VBAR
BBB=3.*SOILNU*RR*RR*DRDD*((ATHTA/AREA)**1.5)*VBAR*VBAR*COST
CCC=CMS*SLRHO*GRAV*ATHTA
FAP=CCC*ADEPTH+AAA+BBB
IF(VBAR.LT.0.000001) GO TO 510
IF(ABS(XFPSD(1,IFP)).LT.0.000001) GO TO 506
SGN=XFPSD(1,IFP)/ABS(XFPSD(1,IFP))

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SOIL1120
SOIL1130
SOIL1140
SOIL1150
SOIL1160
SOIL1170
SOIL1180
SOIL1190
SOIL1200
SOIL1210
SOIL1220
SOIL1230
SOIL1240
SOIL1250
SOIL1260
SOIL1270
SOIL1280
SOIL1290
SOIL1300
SOIL1310
SOIL1320
SOIL1330
SOIL1340
SOIL1350
SOIL1360
SOIL1370
SOIL1380
SOIL1390
SOIL1400
SOIL1410
SOIL1420
SOIL1430
SOIL1440
SOIL1450
SOIL1460
SOIL1470
SOIL1480
SOIL1490
SOIL1500
SOIL1510
SOIL1520
SOIL1530
SOIL1540
SOIL1550
SOIL1560
SOIL1570
SOIL1580
SOIL1590
SOIL1600
SOIL1610
SOIL1620
SOIL1630
SOIL1640
SOIL1650
SOIL1660
SOIL1670

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	GO TO 507	SOIL1680
506	SGN=+1.0	SOIL1690
507	CONTINUE	SOIL1700
	CX=XFPSD(1,IFP)/VBAR	SOIL1710
	IF(VHOR.LT.1.E-10) GO TO 508	SOIL1720
	CY=YFPSD(1,IFP)/VHOR	SOIL1730
	CZ=ZFPSD(1,IFP)/VHOR	SOIL1740
	GO TO 509	SOIL1750
508	CY=0.0	SOIL1760
	CZ=0.0	SOIL1770
509	CONTINUE	SOIL1780
	CH=VHOR/VBAR	SOIL1790
	FH=(CH+CX*ALAM)*FAP	SOIL1800
	SOILX(IFP)=(-CX+SGN*CH*ALAM)*FAP	SOIL1810
	IF(SOILX(IFP).LT.0.)SOILX(IFP)=0.0	SOIL1820
	SOILY(IFP)=-CY*FH	SOIL1830
	SOILZ(IFP)=-CZ*FH	SOIL1840
	GO TO 520	SOIL1850
510	CONTINUE	SOIL1860
	CY=0.0	SOIL1870
	CZ=0.0	SOIL1880
	SOILX(IFP)=FAP	SOIL1890
	SOILY(IFP)=0.0	SOIL1900
	SOILZ(IFP)=0.0	SOIL1910
520	CONTINUE	SOIL1920
	PRESS=SOILX(IFP)/AREA	SOIL1930
C		SOIL1940
C	* * * * *	SOIL1950
C	FOOTPAD ATTENUATION	SOIL1960
C	* * * * *	SOIL1970
C		SOIL1980
1000	CONTINUE	SOIL1990
	IF(FPSTR.GE.ADIST)GO TO 1001	SOIL2000
	IF(PRESS.LE.ATTPRS(3))RETURN	SOIL2010
	ATTH(IFP)=ATTHCK(3)	SOIL2020
	RETURN	SOIL2030
1001	IF(FPSTR.LT.ATTHCK(KJI))GO TO 1002	SOIL2040
	CRPRES=ATTPRS(KJI)	SOIL2050
	GO TO 1004	SOIL2060
1002	IF(KJI.EQ.3)GO TO 1003	SOIL2070
	KJI=KJI+1	SOIL2080
	GO TO 1001	SOIL2090
1003	CRPRES=ATTPRS(3)	SOIL2100
1004	CONTINUE	SOIL2110
	IF(CRPRES.GT.PRESS) RETURN	SOIL2120
	SOILX(IFP)=CRPRES*AREA	SOIL2130
	IF(NTYPE.EQ.0)GO TO 1015	SOIL2140
	ADD=CRPRES/SOILP(2)	SOIL2150
	GO TO 1020	SOIL2160
1015	CONTINUE	SOIL2170
	IF(VBAR.LT.0.000001) GO TO 1016	SOIL2180
	FAP=SOILX(IFP)/(-CX+SGN*CH*ALAM)	SOIL2190
	ADD=(FAP-AAA-BBB)/CCC	SOIL2200
	GO TO 1020	SOIL2210
1016	ADD=SOILX(IFP)/CCC	SOIL2220
1020	CONTINUE	SOIL2230

	IF (ABS(ADD).GT.ADEPTH)ADD=ADEPTH	SOIL2240
	IF (IFIN.EQ.0)ATTH(IFP)=FPSTR+ADD	SOIL2250
	SOILY(IFP)=-CY*SOILX(IFP)	SOIL2260
	SOILZ(IFP)=-CZ*SOILX(IFP)	SOIL2270
	RETURN	SOIL2280
C		SOIL2290
C	FOOTPAD OFF SURFACE	SOIL2300
C		SOIL2310
2000	CONTINUE	SOIL2320
	SOILX(IFP)=0.0	SOIL2330
	SOILY(IFP)=0.0	SOIL2340
	SOILZ(IFP)=0.0	SOIL2350
	RETURN	SOIL2360
	END	SOIL2370

SUBROUTINE INITUP (IYD, CUTVAL, IDIRT)	INIT 10
COMMON / IZZZRM / IPTATL, IPTOTL, LOCNAM(90)	INIT 20
DIMENSION XS(8), IAD(8), IND(8)	INIT 30
COMMON COMINT (400)	INIT 40
EQUIVALENCE (COMINT(329), XS)	INIT 50
EQUIVALENCE (COMINT(344), IAD)	INIT 60
EQUIVALENCE (COMINT(352), IND)	INIT 70
ENTRY LOC	INIT 80
IPTATL = IPTATL + 1	INIT 90
IF (IPTATL .LE. 8) GO TO 3	INIT 100
WRITE (6,1)	INIT 110
1 FORMAT (65H -----JOB TERMINATED, MORE THAN EIGHT CALLS TO LO	INIT 120
*C-----)	INIT 130
STOP	INIT 140
3 CONTINUE	INIT 150
IAD(IPTATL) = IYD	INIT 160
XS(IPTATL) = CUTVAL	INIT 170
IND(IPTATL) = IDIRT	INIT 180
RETURN	INIT 190
ENTRY INUPD	INIT 200
IPTOTL = IPTOTL + 1	INIT 210
IF (IPTOTL .LE. 90) GO TO 4	INIT 220
WRITE (6,2)	INIT 230
2 FORMAT(67H -----JOB TERMINATED, MORE THAN NINETY CALLS TO INUP	INIT 240
*D-----)	INIT 250
STOP	INIT 260
4 LOCNAM(IPTOTL) = IYD	INIT 270
RETURN	INIT 280
END	INIT 290

SUBROUTINE RKCUT	(YD, Y)	RKCT 10
COMMON / IZZRM / IPTATL, IPTOTL, LOCNAM(90)		RKCT 20
DIMENSION X(8), XMN1(8), XS(8), IAD(8), IND(8), Y(4), YD(4)		RKCT 30
COMMON COMINT (400)		RKCT 40
EQUIVALENCE (COMINT(324), T)		RKCT 50
EQUIVALENCE (COMINT(325), HMAX)		RKCT 60
EQUIVALENCE (COMINT(326), HMIN)		RKCT 70
EQUIVALENCE (COMINT(327), EMIN)		RKCT 80
EQUIVALENCE (COMINT(328), EMAX)		RKCT 90
EQUIVALENCE (COMINT(329), XS)		RKCT 100
EQUIVALENCE (COMINT (337), HZ)		RKCT 110
EQUIVALENCE (COMINT(338), CUTERR)		RKCT 120
EQUIVALENCE (COMINT(339), IP)		RKCT 130
EQUIVALENCE (COMINT(340), IVARH)		RKCT 140
EQUIVALENCE (COMINT(341), IMTH)		RKCT 150
EQUIVALENCE (COMINT(342), IPRNT)		RKCT 160
EQUIVALENCE (COMINT(343), IFIN)		RKCT 170
EQUIVALENCE (COMINT(344), IAD)		RKCT 180
EQUIVALENCE (COMINT(352), IND)		RKCT 190
EQUIVALENCE (COMINT(360), J)		RKCT 200
EQUIVALENCE (COMINT(363), IVAL)		RKCT 210
EQUIVALENCE (COMINT(368), H)		RKCT 220
EQUIVALENCE (COMINT(369), XMN1)		RKCT 230
C XMN1 USES COMINT(369-376)		RKCT 240
ENTRY SETUP		RKCT 250
IERROR = 0		RKCT 260
1 IPT2=2**IP		RKCT 270
IPT1=0		RKCT 280
LIST=0		RKCT 290
INDUPD=0		RKCT 300
HZ=HMAX*2.00**(-IP)		RKCT 310
HD2=HZ/2.00		RKCT 320
H=HD2		RKCT 330
IALP=4		RKCT 340
IPRNT=0		RKCT 350
IFIN=0		RKCT 360
IVAL=0		RKCT 370
ISCNT = 0		RKCT 380
IBI1=4		RKCT 390
IBI2=2		RKCT 400
IBU1=1		RKCT 410
IF(IVARH)3,2,3		RKCT 420
2 IBU2=2		RKCT 430
EMAX= ABS(EMAX)		RKCT 440
EMIN= ABS(EMIN)		RKCT 450
RETURN		RKCT 460
3 IBU2=1		RKCT 470
RETURN		RKCT 480
ENTRY INTEG		RKCT 490
GO TO(60,55,55,39),IBI1		RKCT 500
39 Y(4)=Y(1)+H*YD(1)		RKCT 510
IF(IBI2-1)50,41,50		RKCT 520
41 ER= ABS(HZ*(YD(1)-YD(3)))		RKCT 530
IF(Y(1))43,44,43		RKCT 540
43 IF(ER-ER/ ABS(Y(1))) 44,44,4		RKCT 550

4	ER=ER/ ABS(Y(1))	RKCT 560
44	IF(ER-EMAX)45,46,46	RKCT 570
45	IF(ER-EMIN)50,48,48	RKCT 580
48	IVAL=1	RKCT 590
	GO TO 50	RKCT 600
46	IVAL=-8300000	RKCT 610
	IF (HZ .EQ. HMIN) IVAL = 1	RKCT 620
50	YD(3)=YD(1)	RKCT 630
	RETURN	RKCT 640
55	Y(4)=Y(2)+H*YD(1)	RKCT 650
	YD(3)=YD(3)+2.00*YD(1)	RKCT 660
	RETURN	RKCT 670
60	Y(4)=Y(2)+H/6.00*(YD(3)+YD(1))	RKCT 680
	YD(3)=YD(1)	RKCT 690
	RETURN	RKCT 700
	ENTRY UPDAT	RKCT 710
100	IFIN=1	RKCT 720
	IBU3=1	RKCT 730
	IF(IPRNT)110,115,110	RKCT 740
115	IPT1=IPT2	RKCT 750
	IBU3=2	RKCT 760
110	IF(IALP-1)118,120,118	RKCT 770
120	IPT1=IPT1-1	RKCT 780
	IALP=4	RKCT 790
	H=HD2	RKCT 800
	IFIN=0	RKCT 810
121	DO 122 IMVER = 1, IPTOTL	RKCT 820
	KMVER = LOCNAM(IMVER)	RKCT 830
122	COMINT(KMVER) = COMINT(KMVER + 3)	RKCT 840
150	IPRNT=IPT1	RKCT 850
	IBI1=IALP	RKCT 860
	IVAL=0	RKCT 870
	RETURN	RKCT 880
118	IALP=IALP-1	RKCT 890
	GO TO(125,126,127),IALP	RKCT 900
126	IF(IBU2-1)302,301,302	RKCT 910
302	IBI2=1	RKCT 920
	IBU1=2	RKCT 930
301	H=HZ	RKCT 940
	GO TO 121	RKCT 950
125	T=T+HD2	RKCT 960
	GO TO 121	RKCT 970
127	GO TO(130,131),IBU1	RKCT 980
130	DO 132 IMVER = 1, IPTOTL	RKCT 990
	KMVER = LOCNAM(IMVER)	RKCT1000
	COMINT(KMVER + 1) = COMINT(KMVER)	RKCT1010
132	COMINT(KMVER) = COMINT(KMVER + 3)	RKCT1020
	T=T+HD2	RKCT1030
	GO TO 150	RKCT1040
131	IF(IVAL)135,136,135	RKCT1050
136	IF(ISCNT-1)137,137,138	RKCT1060
137	ISCNT=ISCNT+1	RKCT1070
	GO TO 130	RKCT1080
138	HIPT1=IPT1/2	RKCT1090
	XIPT1=IPT1	RKCT1100
	XIPT1=XIPT1/2.00	RKCT1110

	IF(XIPT1-HIPT1)130,140,130	RKCT1120
140	IPT2=IPT2/2	RKCT1130
	IPT1=IPT1/2	RKCT1140
	ISCNT=0	RKCT1150
	IVAL=0	RKCT1160
	H=HZ	RKCT1170
	HD2=HZ	RKCT1180
	HZ=2.00*HZ	RKCT1190
139	IALP=4	RKCT1200
	IBI2=2	RKCT1210
	IBU1=1	RKCT1220
	GO TO 150	RKCT1230
135	ISCNT=0	RKCT1240
	IF(IVAL)160,160,130	RKCT1250
160	IF(IPT1)130,161,161	RKCT1260
161	IF(IBU3-1)163,165,163	RKCT1270
163	IPT1=0	RKCT1280
165	IPT1=2*(IPT1+1)	RKCT1290
	IPT2=2*IPT2	RKCT1300
	T=T-HZ	RKCT1310
	HZ=HD2	RKCT1320
	IF (HZ .LT. HMIN) HZ = HMIN	RKCT1330
	HD2=HZ/2.00	RKCT1340
	H=HD2	RKCT1350
	DO 170 IMVER = 1, IPTOTL	RKCT1360
	KMVER = LOCNAM(IMVER)	RKCT1370
170	COMINT(KMVER) = COMINT(KMVER+1)	RKCT1380
	GO TO 139	RKCT1390
	ENTRY CUT	RKCT1400
	IF(IFIN)200,250,200	RKCT1410
200	J=0	RKCT1420
	IERROR = 1	RKCT1430
	RETURN	RKCT1440
250	K=1	RKCT1450
260	IF (K .LE. IPTATL) GO TO 300	RKCT1460
270	IF(K-1)280,200,280	RKCT1470
280	LL=K-1	RKCT1480
	DO 290 I = 1, LL	RKCT1490
	KMVER = IAD(I)	RKCT1500
290	XMN1(I) = COMINT(KMVER)	RKCT1510
	GO TO 200	RKCT1520
300	XU=XS(K)+.5*CUTERR	RKCT1530
	XL=XS(K)-.5*CUTERR	RKCT1540
	IF(IND(K))500,310,500	RKCT1550
310	KMVER = IAD(K)	RKCT1560
	X(K) = COMINT(KMVER)	RKCT1570
	IF(X(K)-XL) 320,320,400	RKCT1580
320	IF(IVAL)200,330,330	RKCT1590
330	K=K+1	RKCT1600
	IF(K-9)260,280,280	RKCT1610
400	IF(X(K)-XU)410,600,600	RKCT1620
410	J=K	RKCT1630
	IERROR = 1	RKCT1640
	RETURN	RKCT1650
500	KMVER = IAD(K)	RKCT1660
	X(K) = COMINT(KMVER)	RKCT1670

IF(X(K)-XU)510,320,320	RKCT1680
510 IF(X(K)-XL)600,600,410	RKCT1690
600 IF (IVAL .LT. 0) GO TO 200	RKCT1700
IF (IERROR .NE. 0) GO TO 1054	RKCT1710
WRITE(6,1051)K	RKCT1720
1051 FORMAT(1H0,28H***** CUTOFF PASSED BY ,I1,56HTH CUTOFF VARIABL	RKCT1730
1E ON THE INITIAL CALL TO CUT *****)	RKCT1740
STOP	RKCT1750
1054 CONTINUE	RKCT1760
HN=HZ/2.00*((XS(K)-XMN1(K))/(X(K)-XMN1(K)))	RKCT1770
T=T-HZ	RKCT1780
HZ=HN	RKCT1790
HD2=HZ/2.00	RKCT1800
H=HD2	RKCT1810
IALP=4	RKCT1820
IVAL=0	RKCT1830
IBI2=2	RKCT1840
IBU1=1	RKCT1850
IBI1=IALP	RKCT1860
DO 640 IMVER = 1, IPTOTL	RKCT1870
KMVER = LOCNAM(IMVER)	RKCT1880
640 COMINT(KMVER) = COMINT(KMVER+1)	RKCT1890
IFIN=1	RKCT1900
IPT1=IPT2	RKCT1910
ISCNT=0	RKCT1920
J=-1	RKCT1930
IERROR = 1	RKCT1940
RETURN	RKCT1950
END	RKCT1960

SUBROUTINE OUTPUT	OUTPUT	10
DIMENSION DISP(3),VEL(3),ACCEL(3),EL(3),ELD(3),ELDD(3),	OUTPUT	20
* AVEL(9),AACCEL(9),AA(3),BB(3),CC(3),DD(3),EE(3),	OUTPUT	30
* AOUT(3,10),NAC(10)	OUTPUT	40
DIMENSION GC(4,5),GCD(4,5),GCDD(4,5)	OUTPUT	50
DIMENSION XFPS(4,5),YFPS(4,5),ZFPS(4,5),	OUTPUT	60
1 XFPSD(4,5),YFPSD(4,5),ZFPSD(4,5),	OUTPUT	70
2 XFPSDD(4,5),YFPSDD(4,5),ZFPSDD(4,5)	OUTPUT	80
DIMENSION BOUT(75)	OUTPUT	90
COMMON / THISV/ TIMHSA(1), ATTH(5),AM(6,6),AME1(3),INDFPI(6),	OUTPUT	100
1 INDFPC(6),	OUTPUT	110
2 STRPDS(10), STRPMS(5), IPOCDs(10), IPOCMS(5), URCDS (10),	OUTPUT	120
3 URTDS (10), URCMS (5), URTMS (5), SETCDS(10), SETTDS(10),	OUTPUT	130
4 SETCMS(5), SETTMS(5), INDCDS(10), INDTDS(10), INDCMS(10),	OUTPUT	140
5 INDTMS(5), PRFCDS (10), PRFTDS (10), PRFCMS (5), PRFTMS (5),	OUTPUT	150
6 IPRDS (10), IPRMS (5), FRVDSC(10), FRVDST(10), FRVMSC(5),	OUTPUT	160
L FRVMST(5), IPOTDS(10), IPOTMS(5)	OUTPUT	170
EQUIVALENCE (NFTPDS, NOLEG)	OUTPUT	180
COMMON COMINT(400)	OUTPUT	190
EQUIVALENCE (COMINT(3), IBOTM)	OUTPUT	200
EQUIVALENCE (COMINT(4), XSD)	OUTPUT	210
EQUIVALENCE (COMINT(8), XSDD)	OUTPUT	220
EQUIVALENCE (COMINT(12), YS)	OUTPUT	230
EQUIVALENCE (COMINT(16), YSD)	OUTPUT	240
EQUIVALENCE (COMINT(20), YSDD)	OUTPUT	250
EQUIVALENCE (COMINT(24), ZS)	OUTPUT	260
EQUIVALENCE (COMINT(28), ZSD)	OUTPUT	270
EQUIVALENCE (COMINT(32), ZSDD)	OUTPUT	280
EQUIVALENCE (COMINT(36), PHI)	OUTPUT	290
EQUIVALENCE (COMINT(40), PHID)	OUTPUT	300
EQUIVALENCE (COMINT(44), WX)	OUTPUT	310
EQUIVALENCE (COMINT(48), WXD)	OUTPUT	320
EQUIVALENCE (COMINT(52), THTA)	OUTPUT	330
EQUIVALENCE (COMINT(56), THTAD)	OUTPUT	340
EQUIVALENCE (COMINT(60), WY)	OUTPUT	350
EQUIVALENCE (COMINT(64), WYD)	OUTPUT	360
EQUIVALENCE (COMINT(68), PSI)	OUTPUT	370
EQUIVALENCE (COMINT(72), PSID)	OUTPUT	380
EQUIVALENCE (COMINT(76), WZ)	OUTPUT	390
EQUIVALENCE (COMINT(80), WZD)	OUTPUT	400
EQUIVALENCE (COMINT(84), GC)	OUTPUT	410
EQUIVALENCE (COMINT(104), GCD)	OUTPUT	420
EQUIVALENCE (COMINT(124), GCDD)	OUTPUT	430
EQUIVALENCE (COMINT(144), XFPS)	OUTPUT	440
EQUIVALENCE (COMINT(164), XFPSD)	OUTPUT	450
EQUIVALENCE (COMINT(184), XFPSDD)	OUTPUT	460
EQUIVALENCE (COMINT(204), YFPS)	OUTPUT	470
EQUIVALENCE (COMINT(224), YFPSD)	OUTPUT	480
EQUIVALENCE (COMINT(244), YFPSDD)	OUTPUT	490
EQUIVALENCE (COMINT(264), ZFPS)	OUTPUT	500
EQUIVALENCE (COMINT(284), ZFPSD)	OUTPUT	510
EQUIVALENCE (COMINT(304), ZFPSDD)	OUTPUT	520
EQUIVALENCE (COMINT(324), TIME)	OUTPUT	530
EQUIVALENCE (COMINT(325), HMAX)	OUTPUT	540
EQUIVALENCE (COMINT(326), HMIN)	OUTPUT	550

EQUIVALENCE (COMINT(327), EMIN)	OUTPUT 560
EQUIVALENCE (COMINT(328), EMAX)	OUTPUT 570
EQUIVALENCE (COMINT(329), XSI)	OUTPUT 580
EQUIVALENCE (COMINT (337), HZ)	OUTPUT 590
EQUIVALENCE (COMINT(338), CUTERR)	OUTPUT 600
EQUIVALENCE (COMINT(339), IP)	OUTPUT 610
EQUIVALENCE (COMINT(340), IVARH)	OUTPUT 620
EQUIVALENCE (COMINT(341), IMTH)	OUTPUT 630
EQUIVALENCE (COMINT(342), IPRNT)	OUTPUT 640
EQUIVALENCE (COMINT(343), IFIN)	OUTPUT 650
EQUIVALENCE (COMINT(344), IAD)	OUTPUT 660
EQUIVALENCE (COMINT(352), IND)	OUTPUT 670
EQUIVALENCE (COMINT(360), JCUT)	OUTPUT 680
EQUIVALENCE (COMINT(361), IPTCNT)	OUTPUT 690
EQUIVALENCE (COMINT(364), XS)	OUTPUT 700
C COMMON COMINT(364-367) USED BY XS	OUTPUT 710
COMMON	OUTPUT 720
1 CBMASS, CBIXX , CBIXZ , CBIYY , CBIYZ , CBIZZ , FPMASS,CBIXY,	OUTPUT 730
2 DC(3,3) , XFP(5), YFP(5), ZFP(5), WNX(5), WNY(5),	OUTPUT 740
3 WNZ(5), PX(5) , PY(5) , PZ(5) , GM(5) , OMEGA(5)	OUTPUT 750
4 GRAV , GRAVE , ZETA , FTS (6) , FSXSI(5)	OUTPUT 760
5 FSYSI(5) , FSZSI(5) , SOILX(5)	OUTPUT 770
6 SOILY(5) , SOILZ(5) , PMSX(5,5)	OUTPUT 780
7 PMSY(5,5) , PMSZ(5,5) , PDSX(10,5)	OUTPUT 790
8 PDSY(10,5) , PDSZ(10,5) , FLXS , FLYS , FLZS ,	OUTPUT 800
9 TLXL , TLYL , TLZL , SLO , XMSCB(5)	OUTPUT 810
C YMSCB(5) , ZMSCB(5) , XDSCB(10)	OUTPUT 820
D YDSCB(10) , ZDSCB(10) , ILEG , IMS	OUTPUT 830
E FSTX , FSTY , FSTZ , PVCBX , PVCBY , PVCBZ ,	OUTPUT 840
F PVFPX , PVFPY , PVFPZ , NOLEG , SLOMS(5) ,	OUTPUT 850
G SLODS(10)	OUTPUT 860
COMMON	OUTPUT 870
1 PFCMS(5) , PFCDS(5) , PFTDS(5)	OUTPUT 880
2 PFTMS(5) , SRCDS(5) , SRCMS(5)	OUTPUT 890
3 SRTDS(5) , SRTMS(5) , COEFDS, COEFMS, GAMDS ,	OUTPUT 900
4 GAMMS , SRUCDS, SRUCMS, SRUTDS, SRUTMS, SCMXDS, SCMXMS,	OUTPUT 910
5 STMXDS, STMXMS, CDCDS(5) , CDCMS(5)	OUTPUT 920
6 CDTDS(5) , CDTMS(5) , FRICDS, FRICMS, IRETDS,	OUTPUT 930
7 IRETMS, STROKE(10) , STRKDS(10) , STRKMS(5)	OUTPUT 940
8 LENGTH,CDXS,CDYS,CDZS , NTYPE , RAD(4), SS(4) , ATTHCK(3),	OUTPUT 950
9 ATTPRS(3) , ADIST , SOILP(3) , NMODES,	OUTPUT 960
A COEF , GAMMA , FRIC , SCMAX , STMAX , CDC(5), CDT(5),	OUTPUT 970
B SRULT , SRULC , SRT(5), SRC(5), PFT(5), PFC(5), GFLEGS(5),	OUTPUT 980
C CURMSV, CURMSL, INDFXD, INDFYD, INDFZD, INDFXR, INDFYR,	OUTPUT 990
D INDFZR, TIMAX , DRAGST, IFP	OUTPUT1000
COMMON	OUTPUT1010
1CMS,CDCONT,SOILNU,SLRHO, NOOUT,XOUT(10),YOUT(10),ZOUT(10),	OUTPUT1020
2MODEIN,POUTX(10,5),POUTY(10,5),POUTZ(10,5),PCGX(5),PCGY(5),	OUTPUT1030
3PCGZ(5),AEI1,AEI2,FORMS(5),FORDS(10),FORCE,NFORC,SAVMSX(5),	OUTPUT1040
4SAVMSZ(5),SAVDSX(10),SAVDSY(10),SAVDSZ(10),IQUOUT,GSINZT,	OUTPUT1050
5GCOSZT,STAB,STABVL,ISTAB,JCKSAB,VELX,VELY,VELZ,SAVMSY(5)	OUTPUT1060
COMMON	OUTPUT1070
1 SMXMSC(5), TMXMSC(5), SMXMST(5), TMXMST(5),	OUTPUT1080
2 SMXDSC(10), TMXDSC(10), SMXDST(10), TMXDST(10)	OUTPUT1090
3 ,SLNGMS(5),SLNGDS(10), CURDSL, INLEG , IFPPRT,	OUTPUT1100
4 IMPACT(5),IPRTFP(5), KOUNT(5) ,ANGX, ANGY, ANGZ	OUTPUT1110

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RADIAN=57.295779513
DUM1=PHI*RADIAN
DUM2=WX*RADIAN
DUM3=WXD*RADIAN
DUM4=THTA*RADIAN
DUM5=WY*RADIAN
DUM6=WYD*RADIAN
DUM7=PSI*RADIAN
DUM8=WZ*RADIAN
DUM9=WZD*RADIAN
AVEL(1)=0.
AVEL(2)=WZ
AVEL(3)=-WY
AVEL(4)=-WZ
AVEL(5)=0.
AVEL(6)=WX
AVEL(7)=WY
AVEL(8)=-WX
AVEL(9)=0.
AACCEL(1)=0.
AACCEL(2)=WZD
AACCEL(3)=-WYD
AACCEL(4)=-WZD
AACCEL(5)=0.
AACCEL(6)=WXD
AACCEL(7)=WYD
AACCEL(8)=-WXD
AACCEL(9)=0.
DISP(1)=XS
DISP(2)=YS
DISP(3)=ZS
VEL(1)=XSD
VEL(2)=YSD
VEL(3)=ZSD
ACCEL(1)=XSDD
ACCEL(2)=YSDD
ACCEL(3)=ZSDD

```

C
C
C

CENTER OF GRAVITY MOTIONS

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WRITE(6,350)
WRITE(6,1000)TIME,HZ
IF(NMODES.EQ.0)GO TO 90
DO 50 I=1,3
EL(I)=0.
ELD(I)=0.
50 ELDD(I)=0.
DO 51 I=1,NMODES
EL(1)=EL(1)+PCGX(I)*GC(1,I)
EL(2)=EL(2)+PCGY(I)*GC(1,I)
EL(3)=EL(3)+PCGZ(I)*GC(1,I)
ELD(1)=ELD(1)+PCGX(I)*GCD(1,I)
ELD(2)=ELD(2)+PCGY(I)*GCD(1,I)
ELD(3)=ELD(3)+PCGZ(I)*GCD(1,I)
ELDD(1)=ELDD(1)+PCGX(I)*GCDD(1,I)
ELDD(2)=ELDD(2)+PCGY(I)*GCDD(1,I)

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OUTP1120
OUTP1130
OUTP1140
OUTP1150
OUTP1160
OUTP1170
OUTP1180
OUTP1190
OUTP1200
OUTP1210
OUTP1220
OUTP1230
OUTP1240
OUTP1250
OUTP1260
OUTP1270
OUTP1280
OUTP1290
OUTP1300
OUTP1310
OUTP1320
OUTP1330
OUTP1340
OUTP1350
OUTP1360
OUTP1370
OUTP1380
OUTP1390
OUTP1400
OUTP1410
OUTP1420
OUTP1430
OUTP1440
OUTP1450
OUTP1460
OUTP1470
OUTP1480
OUTP1490
OUTP1500
OUTP1510
OUTP1520
OUTP1530
OUTP1540
OUTP1550
OUTP1560
OUTP1570
OUTP1580
OUTP1590
OUTP1600
OUTP1610
OUTP1620
OUTP1630
OUTP1640
OUTP1650
OUTP1660
OUTP1670

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	ELDD(3)=ELDD(3)+PCGZ(I)*GCDD(1,I)	OUTP1680
51	CONTINUE	OUTP1690
	CALL GMPRD(DC,EL,AA,3,3,1)	OUTP1700
	DO 52 I=1,3	OUTP1710
52	DISP(I)=DISP(I)+AA(I)	OUTP1720
	CALL GMPRD(AVEL,EL,AA,3,3,1)	OUTP1730
	DO 53 I=1,3	OUTP1740
53	BB(I)=ELD(I)+AA(I)	OUTP1750
	CALL GMPRD(DC,BB,CC,3,3,1)	OUTP1760
	DO 54 I=1,3	OUTP1770
54	VEL(I)=VEL(I)+CC(I)	OUTP1780
	CALL GMPRD(AVEL,AA,BB,3,3,1)	OUTP1790
	CALL GMPRD(AVEL,ELD,AA,3,3,1)	OUTP1800
	CALL GMPRD(AACCEL,EL,CC,3,3,1)	OUTP1810
	DO 55 I=1,3	OUTP1820
55	DD(I)=ELDD(I)+CC(I)+2.*AA(I)+BB(I)	OUTP1830
	CALL GMPRD(DC,DD,AA,3,3,1)	OUTP1840
	DO 56 I=1,3	OUTP1850
56	ACCEL(I)=ACCEL(I)+AA(I)	OUTP1860
90	CONTINUE	OUTP1870
	WRITE(6,1001)DISP(1),VEL(1),ACCEL(1),DUM1,DUM2,DUM3,	OUTP1880
	* DISP(2),VEL(2),ACCEL(2),DUM4,DUM5,DUM6,	OUTP1890
	* DISP(3),VEL(3),ACCEL(3),DUM7,DUM8,DUM9	OUTP1900
	WRITE(6,1022)	OUTP1910
	DO 100 I=1,NFTPDS	OUTP1920
	IF(INDFPC(I).LE.0)GO TO 100	OUTP1930
	WRITE(6,1003)I,	OUTP1940
	* XFPS(1,I),XFPSD(1,I),XFPSDD(1,I),	OUTP1950
	* YFPS(1,I),YFPSD(1,I),YFPSDD(1,I),	OUTP1960
	* ZFPS(1,I),ZFPSD(1,I),ZFPSDD(1,I)	OUTP1970
100	CONTINUE	OUTP1980
C		OUTP1990
C	ACCELERATION AND INTEGRATED QUANTITIES	OUTP2000
C		OUTP2010
	DO 60 I=1,3	OUTP2020
60	AA(I)=(DC(1,I)*ACCEL(1)+DC(2,I)*ACCEL(2)+DC(3,I)*ACCEL(3))/GRAVE	OUTP2030
	IF(NOOUT.EQ.0)GO TO 202	OUTP2040
	DO 200 J=1,NOOUT	OUTP2050
	DO 102 I=1,3	OUTP2060
	ELD(I)=0.	OUTP2070
102	ELDD(I)=0.	OUTP2080
	EL(1)=XOUT(J)	OUTP2090
	EL(2)=YOUT(J)	OUTP2100
	EL(3)=ZOUT(J)	OUTP2110
	IF(NMODES.EQ.0)GO TO 111	OUTP2120
	DO 110 I=1,NMODES	OUTP2130
	EL(1)=EL(1)+POUTX(J,I)*GC(1,I)	OUTP2140
	EL(2)=EL(2)+POUTY(J,I)*GC(1,I)	OUTP2150
	EL(3)=EL(3)+POUTZ(J,I)*GC(1,I)	OUTP2160
	ELD(1)=ELD(1)+POUTX(J,I)*GCD(1,I)	OUTP2170
	ELD(2)=ELD(2)+POUTY(J,I)*GCD(1,I)	OUTP2180
	ELD(3)=ELD(3)+POUTZ(J,I)*GCD(1,I)	OUTP2190
	ELDD(1)=ELDD(1)+POUTX(J,I)*GCDD(1,I)	OUTP2200
	ELDD(2)=ELDD(2)+POUTY(J,I)*GCDD(1,I)	OUTP2210
	ELDD(3)=ELDD(3)+POUTZ(J,I)*GCDD(1,I)	OUTP2220
110	CONTINUE	OUTP2230

111	CONTINUE	OUTP2240
	CALL GMPRD(AVEL,EL,BB,3,3,1)	OUTP2250
	CALL GMPRD(AVEL,BB,CC,3,3,1)	OUTP2260
	CALL GMPRD(AACCEL,EL,BB,3,3,1)	OUTP2270
	CALL GMPRD(AVEL,ELD,DD,3,3,1)	OUTP2280
	DO 120 I=1,3	OUTP2290
120	EE(I)=ELDD(I)+BB(I)+2.*DD(I)+CC(I)	OUTP2300
	AOUT(1,J)=(DC(1,1)*XSDD+DC(2,1)*YSDD+DC(3,1)*ZSDD+EE(1))/GRAVE	OUTP2310
	AOUT(2,J)=(DC(1,2)*XSDD+DC(2,2)*YSDD+DC(3,2)*ZSDD+EE(2))/GRAVE	OUTP2320
	AOUT(3,J)=(DC(1,3)*XSDD+DC(2,3)*YSDD+DC(3,3)*ZSDD+EE(3))/GRAVE	OUTP2330
200	CONTINUE	OUTP2340
202	CONTINUE	OUTP2350
	IF(NMODES.EQ.0)GO TO 220	OUTP2360
	IF(IQUOUT.EQ.0)GO TO 220	OUTP2370
	IF(NOOUT.NE.0)GO TO 210	OUTP2380
C	PRINT C.G. ACCELERATION AND INTEGRATED QUANTITIES	OUTP2390
	WRITE(6,1005)	OUTP2400
	WRITE(6,1007)AA(1),AA(2),AA(3),XS,XSD,XSDD	OUTP2410
	WRITE(6,1008)YS,YSD,YSD	OUTP2420
	WRITE(6,1009)ZS,ZSD,ZSDD	OUTP2430
	DO 201 I=1,NMODES	OUTP2440
201	WRITE(6,1010)I,GC(1,I),GCD(1,I),GCDD(1,I)	OUTP2450
	GO TO 300	OUTP2460
C	PRINT C.G. ACCELERATION, SECONDARY POINTS, AND INTEGRATED QUANT.	OUTP2470
210	CONTINUE	OUTP2480
	DO 95 I=1,10	OUTP2490
95	NAC(I)=I	OUTP2500
	WRITE(6,1005)	OUTP2505
	WRITE(6,1007)AA(1),AA(2),AA(3),XS,XSD,XSDD	OUTP2510
	WRITE(6,1011)NAC(1),AOUT(1,1),AOUT(2,1),AOUT(3,1),YS,YSD,YSD	OUTP2520
	WRITE(6,1012)NAC(2),AOUT(1,2),AOUT(2,2),AOUT(3,2),ZS,ZSD,ZSDD	OUTP2530
	DO 211 I=1,NMODES	OUTP2540
	II=I+2	OUTP2550
211	WRITE(6,1013)NAC(II),AOUT(1,II),AOUT(2,II),AOUT(3,II),I,GC(1,I),	OUTP2560
	* GCD(1,I),GCDD(1,I)	OUTP2570
	IF(II.GE.NOOUT)GO TO 300	OUTP2580
	II=II+1	OUTP2590
	DO 212 I=II,NOOUT	OUTP2600
212	WRITE(6,1014)NAC(I),AOUT(1,I),AOUT(2,I),AOUT(3,I)	OUTP2610
	GO TO 300	OUTP2620
C	PRINT C.G. ACCELERATIONS AND SECONDARY POINTS	OUTP2630
220	CONTINUE	OUTP2640
	WRITE(6,1004)	OUTP2650
	WRITE(6,1006)AA(1),AA(2),AA(3)	OUTP2660
	IF(NOOUT.EQ.0)GO TO 300	OUTP2670
	DO 96 I=1,10	OUTP2675
96	NAC(I)=I	OUTP2676
	DO 221 I=1,NOOUT	OUTP2680
221	WRITE(6,1014)NAC(I),AOUT(1,I),AOUT(2,I),AOUT(3,I)	OUTP2690
C		OUTP2700
C	STRUT INFORMATION	OUTP2710
C		OUTP2720
300	CONTINUE	OUTP2730
	WRITE(6,1020)	OUTP2740
	DO 205 I=1,NFTPDS	OUTP2750
	II=2*(I-1)+1	OUTP2760

	II2=II+1	OUTPUT2770
	CRUSH=SS(1)-ATTH(I)	OUTPUT2780
205	WRITE(6,1021)I,CRUSH ,FORMS(I),STRKMS(I),II,FORDS(II),STRKDS(II),	OUTPUT2790
	*II2,FORDS(II2),STRKDS(II2)	OUTPUT2800
C		OUTPUT2810
C	CHECK LANDER STABILITY	OUTPUT2820
C		OUTPUT2830
	IF(JCKSAB.EQ.0)GO TO 400	OUTPUT2840
C		OUTPUT2850
	CALL STABLE	OUTPUT2860
C		OUTPUT2870
	IF(ISTAB.NE.0)GO TO 400	OUTPUT2880
	WRITE(6,1015)STAB,STABVL	OUTPUT2890
400	CONTINUE	OUTPUT2900
	IF(NFORC.EQ.0) RETURN	OUTPUT2910
C		OUTPUT2920
C	WRITE TIME HISTORY QUANTITIES ON TAPE FOR USE WITH LANDING	OUTPUT2930
C	LOADS PROGRAM	OUTPUT2940
C	1. NFORC=1 RESULTS IN TIME HISTORY TAPE BEING GENERATED	OUTPUT2950
C	2. TIME HISTORY INFORMATION WRITTEN ON TAPE3	OUTPUT2960
C		OUTPUT2970
	BOUT(1)=TIME	OUTPUT2980
	BOUT(2)=HMAX	OUTPUT2990
	BOUT(3)=XSDD	OUTPUT3000
	BOUT(4)=YSDD	OUTPUT3010
	BOUT(5)=ZSDD	OUTPUT3020
	BOUT(6)=PHI	OUTPUT3030
	BOUT(7)=THTA	OUTPUT3040
	BOUT(8)=PSI	OUTPUT3050
	BOUT(9)=WX	OUTPUT3060
	BOUT(10)=WY	OUTPUT3070
	BOUT(11)=WZ	OUTPUT3080
	BOUT(12)=WXD	OUTPUT3090
	BOUT(13)=WYD	OUTPUT3100
	BOUT(14)=WZD	OUTPUT3110
	NSP=14	OUTPUT3120
	DO 450 I=1,NMODES	OUTPUT3130
	NNL=NSP+3*I	OUTPUT3140
	BOUT(NNL-2)=GC(1,I)	OUTPUT3150
	BOUT(NNL-1)=GCD(1,I)	OUTPUT3160
450	BOUT(NNL)=GCDD(1,I)	OUTPUT3170
	NSP=14+3*NMODES	OUTPUT3180
	DO 451 I=1,NOLEG	OUTPUT3190
	NNL=NSP+3*I	OUTPUT3200
	BOUT(NNL-2)=SAVMSX(I)	OUTPUT3210
	BOUT(NNL-1)=SAVMSY(I)	OUTPUT3220
451	BOUT(NNL)=SAVMSZ(I)	OUTPUT3230
	II=2*NOLEG	OUTPUT3240
	NSP=NSP+3*NOLEG	OUTPUT3250
	DO 452 I=1,II	OUTPUT3260
	NNL=NSP+3*I	OUTPUT3270
	BOUT(NNL-2)=SAVDSX(I)	OUTPUT3280
	BOUT(NNL-1)=SAVDSY(I)	OUTPUT3290
452	BOUT(NNL)=SAVDSZ(I)	OUTPUT3300
	NSP=NSP+3*II	OUTPUT3310
	WRITE(3)(BOUT(I),I=1,NSP)	OUTPUT3320


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1007 FORMAT(6X,18HCENTER OF GRAVITY ,3(1X,E10.3) ,26X,13HCENTER BODY X,OUTP3890
      *2X,3(1X,E10.3))                                OUTP3900
1008 FORMAT(83X,13HCENTER BODY Y,2X,3(1X,E10.3))    OUTP3910
1009 FORMAT(83X,13HCENTER BODY Z,2X,3(1X,E10.3))    OUTP3920
1010 FORMAT(83X,4HMODE,I4,7X,3(1X,E10.3))           OUTP3930
1011 FORMAT(6X,6HPOINT ,I2,10X,3(1X,E10.3),26X,13HCENTER BODY Y,2X,
      *3(1X,E10.3))                                    OUTP3940
1012 FORMAT(6X,6HPOINT ,I2,10X,3(1X,E10.3),26X,13HCENTER BODY Z,2X,
      *3(1X,E10.3))                                    OUTP3960
1013 FORMAT(6X,6HPOINT ,I2,10X,3(1X,E10.3),26X,4HMODE,I4,7X,3(1X,
      *E10.3))                                          OUTP3970
1014 FORMAT(6X,6HPOINT ,I2,10X,3(1X,E10.3))          OUTP3980
1015 FORMAT(/3X,16HLANDER STABILITY/6X,20HSTABILITY ANGLE = ,
      *E10.3/6X,20HPITCHING VELOCITY = ,E10.3)       OUTP3990
1020 FORMAT(/3X,30HLANDING GEAR STRUT INFORMATION/6X,3HLEG,6X,
      *5HCRUSH,18X,4HLOAD,                             OUTP4000
      *6X,6HSTROKE,20X,4HLOAD,6X,6HSTROKE,20X,4HLOAD,6X,6HSTROKE)
1021 FORMAT(7X,I1,4X,E10.3,2X,10HMAIN STRUT,2(1X,E10.3),12H DRAG STRUT
      *,I2,                                             OUTP4010
      *2(1X,E10.3),12H DRAG STRUT,I2,2(1X,E10.3))    OUTP4020
1022 FORMAT(/3X,43HFOOTPAD MOTIONS - SURFACE COORDINATE SYSTEM
      */47X,1HX,33X,1HY,                                OUTP4030
      *33X,1HZ/34X,4HDISP,8X,3HVEL,7X,5HACCEL,7X,4HDISP,8X,3HVEL,7X,
      *5HACCEL,7X,4HDISP,8X,3HVEL,7X,5HACCEL)         OUTP4040
2000 FORMAT(/3X,19HSUMMARY INFORMATION/)              OUTP4050
2001 FORMAT(7X,I2,2X,2(2X,E10.3),2X,2(2X,E10.3))    OUTP4060
2003 FORMAT(/,6X,24HEND OF CASE - TIME LIMIT)        OUTP4070
2004 FORMAT(/,6X,31HEND OF CASE - PITCH INSTABILITY) OUTP4080
2005 FORMAT(/,6X,29HEND OF CASE - YAW INSTABILITY)   OUTP4090
2006 FORMAT(3X,26HMAXIMUM MAIN STRUT STROKES,/
      *19X,11HCOMPRESSION,17X,7HTENSION/6X,5HSTRUT,5X,4HTIME,7X,
      *6HSTROKE,9X,4HTIME,7X,6HSTROKE)               OUTP4100
2007 FORMAT(3X,26HMAXIMUM DRAG STRUT STROKES,/
      *19X,11HCOMPRESSION,17X,7HTENSION/6X,5HSTRUT,5X,4HTIME,7X,
      *6HSTROKE,9X,4HTIME,7X,6HSTROKE)               OUTP4110
6200 FORMAT(/10X,35HMAIN STRUT BOTTOMED ON TENSION SIDE,/
      * 30X,8HSTRUT = ,I2)                             OUTP4120
6201 FORMAT(/10X,39HMAIN STRUT BOTTOMED ON COMPRESSION SIDE,/
      * 30X,8HSTRUT = ,I2)                             OUTP4130
6202 FORMAT(/10X,35HDRAG STRUT BOTTOMED ON TENSION SIDE,/
      * 30X,8HSTRUT = ,I2)                             OUTP4140
6203 FORMAT(/10X,39HDRAG STRUT BOTTOMED ON COMPRESSION SIDE,/
      * 30X,8HSTRUT = ,I2)                             OUTP4150
      END                                              OUTP4160
      OUTP4170
      OUTP4180
      OUTP4190
      OUTP4200
      OUTP4210
      OUTP4220
      OUTP4230
      OUTP4240
      OUTP4250
      OUTP4260
      OUTP4270
      OUTP4280
      OUTP4290
      OUTP4300
      OUTP4310
      OUTP4320

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SUBROUTINE STABLE
C
C THIS ROUTINE CHECKS THE STABILITY OF THE LANDER
C THE FOLLOWING FLAG IS RETURNED
C 1. ISTAB=0 - LANDER STABLE
C 2. ISTAB=1 - PITCH INSTABILITY
C 3. ISTAB=2 - YAW INSTABILITY
C
C DIMENSION DX(5),DY(5),DZ(5),SGN(5),ANG(5)
C
C DIMENSION GC(4,5), GCD(4,5), GCDD(4,5)
C
C DIMENSION XFPS(4,5), YFPS(4,5), ZFPS(4,5),
1 XFPSD(4,5), YFPSD(4,5), ZFPSD(4,5),
2 XFPSDD(4,5), YFPSDD(4,5), ZFPSDD(4,5)
C
COMMON / THISV/ TIMHSA(1), ATTH(5),AM(6,6),AME1(3),INDFPI(6),
1 INDFPC(6),
2 STRPDS(10), STRPMS( 5), IPOCDs(10), IPOCMS( 5), URCDS (10),
3 URTDS (10), URCMS ( 5), URTMS ( 5), SETCDS(10), SETTDS(10),
4 SETCMS( 5), SETTMS( 5), INDCDS(10), INDTDS(10), INDCMS(10),
5 INDTMS( 5), PRFCDS (10), PRFTDS (10), PRFCMS ( 5), PRFTMS ( 5),
6 IPRDS (10), IPRMS ( 5), FRVDSC(10), FRVDST(10), FRVMSC( 5),
L FRVMST( 5), IPOTDS(10), IPOTMS( 5)
EQUIVALENCE ( NFTPDS, NOLEG )
COMMON COMINT(400)
EQUIVALENCE ( COMINT( 4 ), XSD )
EQUIVALENCE ( COMINT( 8 ), XSDD )
EQUIVALENCE ( COMINT( 12 ), YS )
EQUIVALENCE ( COMINT( 16 ), YSD )
EQUIVALENCE ( COMINT( 20 ), YSDD )
EQUIVALENCE ( COMINT( 24 ), ZS )
EQUIVALENCE ( COMINT( 28 ), ZSD )
EQUIVALENCE ( COMINT( 32 ), ZSDD )
EQUIVALENCE ( COMINT( 36 ), PHI )
EQUIVALENCE ( COMINT( 40 ), PHID )
EQUIVALENCE ( COMINT( 44 ), WX )
EQUIVALENCE ( COMINT( 48 ), WXD )
EQUIVALENCE ( COMINT( 52 ), THTA )
EQUIVALENCE ( COMINT( 56 ), THTAD )
EQUIVALENCE ( COMINT( 60 ), WY )
EQUIVALENCE ( COMINT( 64 ), WYD )
EQUIVALENCE ( COMINT( 68 ), PSI )
EQUIVALENCE ( COMINT( 72 ), PSID )
EQUIVALENCE ( COMINT( 76 ), WZ )
EQUIVALENCE ( COMINT( 80 ), WZD )
EQUIVALENCE ( COMINT( 84 ), GC )
EQUIVALENCE ( COMINT(104), GCD )
EQUIVALENCE ( COMINT(124), GCDD )
EQUIVALENCE ( COMINT(144), XFPS )
EQUIVALENCE ( COMINT(164), XFPSD )
EQUIVALENCE ( COMINT(184), XFPSDD )
EQUIVALENCE ( COMINT(204), YFPS )
EQUIVALENCE ( COMINT(264), ZFPS )
EQUIVALENCE ( COMINT(224), YFPSD )
STAB 10
STAB 20
STAB 30
STAB 40
STAB 50
STAB 60
STAB 70
STAB 80
STAB 90
STAB 100
STAB 110
STAB 120
STAB 130
STAB 140
STAB 150
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STAB 170
STAB 180
STAB 190
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STAB 220
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STAB 520
STAB 530
STAB 540
STAB 550

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EQUIVALENCE (COMINT(244), YFPSDD)	STAB 560
EQUIVALENCE (COMINT(284), ZFPSD)	STAB 570
EQUIVALENCE (COMINT(304), ZFPSDD)	STAB 580
EQUIVALENCE (COMINT(324), TIME)	STAB 590
EQUIVALENCE (COMINT(325), HMAX)	STAB 600
EQUIVALENCE (COMINT(326), HMIN)	STAB 610
EQUIVALENCE (COMINT(327), EMIN)	STAB 620
EQUIVALENCE (COMINT(328), EMAX)	STAB 630
EQUIVALENCE (COMINT(329), XSI)	STAB 640
EQUIVALENCE (COMINT (337), HZ)	STAB 650
EQUIVALENCE (COMINT(338), CUTERR)	STAB 660
EQUIVALENCE (COMINT(339), IP)	STAB 670
EQUIVALENCE (COMINT(340), IVARH)	STAB 680
EQUIVALENCE (COMINT(341), IMTH)	STAB 690
EQUIVALENCE (COMINT(342), IPRNT)	STAB 700
EQUIVALENCE (COMINT(343), IFIN)	STAB 710
EQUIVALENCE (COMINT(344), IAD)	STAB 720
EQUIVALENCE (COMINT(352), IND)	STAB 730
EQUIVALENCE (COMINT(360), JCUT)	STAB 740
EQUIVALENCE (COMINT(361), IPTCNT)	STAB 750
EQUIVALENCE (COMINT (362), IDSETN)	STAB 760
EQUIVALENCE (COMINT(363), IVAL)	STAB 770
EQUIVALENCE (COMINT(364), XS)	STAB 780
COMINT(364-367) USED BY XS	STAB 790
C	
COMMON	STAB 800
1 CBMASS, CBIXX , CBIXZ , CBIYY , CBIYZ , CBIZZ , FPMASS,CBIXY,	STAB 810
2 DC(3,3) , XFP(5), YFP(5), ZFP(5), WNX(5), WNY(5),	STAB 820
3 WNZ(5), PX(5) , PY(5) , PZ(5) , GM(5) , OMEGA(5) ,	STAB 830
4 GRAV , GRAVE , ZETA , FTS (6) , FSXSI(5) ,	STAB 840
5 FSYSI(5) , FSZSI(5) , SOILX(5) ,	STAB 850
6 SOILY(5) , SOILZ(5) , PMSX(5,5) ,	STAB 860
7 PMSY(5,5) , PMSZ(5,5) , PDSX(10,5) ,	STAB 870
8 PDSY(10,5) , PDSZ(10,5) , FLXS , FLYS , FLZS ,	STAB 880
9 TLXL , TLYL , TLZL , SLO , XMSCB(5) ,	STAB 890
C YMSCB(5) , ZMSCB(5) , XDSCB(10) ,	STAB 900
D YDSCB(10) , ZDSCB(10) , ILEG , IMS ,	STAB 910
E FSTX , FSTY , FSTZ , PVCBX , PVCBY , PVCBZ ,	STAB 920
F PVFPX , PVFPY , PVFPZ , NOLEG , SLOMS(5) ,	STAB 930
G SLODS(10)	STAB 940
COMMON	STAB 950
1 PFCMS(5) , PFCDS(5) , PFTDS(5) ,	STAB 960
2 PFTMS(5) , SRCDS(5) , SRCMS(5) ,	STAB 970
3 SRTDS(5) , SRTMS(5) , COEFDS, COEFMS, GAMDS ,	STAB 980
4 GAMMS , SRUCDS, SRUCMS, SRUTDS, SRUTMS, SCMXDS, SCMXMS, ,	STAB 990
5 STMXDS, STMXMS, CDCDS(5) , CDCMS(5) ,	STAB1000
6 CDTDS(5) , CDTMS(5) , FRICDS, FRICMS, IRETDS, ,	STAB1010
7 IRETMS, STROKE(10) , STRKDS(10) , STRKMS(5) ,	STAB1020
8 LENGTH,CDXS,CDYS,CDZS , NTYPE , RAD(4), SS(4) , ATTHCK(3), ,	STAB1030
9 ATTPRS(3) , ADIST , SOILP(3) , NMODES, ,	STAB1040
A COEF , GAMMA , FRIC , SCMAX , STMAX , CDC(5), CDT(5), ,	STAB1050
B SRULT , SRULC , SRT(5), SRC(5), PFT(5), PFC(5), GFLEGS(5), ,	STAB1060
C CURMSV, CURMSL, INDFXD, INDFYD, INDFZD, INDFXR, INDFYR, ,	STAB1070
D INDFZR, TIMAX , DRAGST, IFP	STAB1080
C	STAB1090
COMMON	STAB1100
1CMS,CDCONT,SOILNU,SLRHO, NOOUT,XOUT(10),YOUT(10),ZOUT(10),	STAB1110

	2MODEIN,POUTX(10,5),POUTY(10,5),POUTZ(10,5),PCGX(5),PCGY(5),	STAB1120
	3PCGZ(5),AEI1,AEI2,FORMS(5),FORDS(10),FORCE,NFORC,SAVMSX(5),	STAB1130
	4SAVMSZ(5),SAVDSX(10),SAVDSY(10),SAVDSZ(10),IQUOUT,GSINZT,	STAB1140
	5GCOSZT,STAB,STABVL,ISTAB,JCKSAB,VELX,VELY,VELZ,SAVMSY(5)	STAB1150
	COMMON	STAB1160
	1 SMXMSC(5), TMXMSC(5), SMXMST(5), TMXMST(5),	STAB1170
	2 SMXDSC(10), TMXDSC(10), SMXDST(10), TMXDST(10)	STAB1180
	3 ,SLNGMS(5),SLNGDS(10),CURDSL,INLEG,IFPRT,	STAB1190
	4 IMPACT(5),IPRTFP(5),KOUNT(5),ANGX,ANGY,ANGZ	STAB1200
C		STAB1210
C	SUBROUTINE INITIALIZATION	STAB1220
C		STAB1230
	DO 50 I=1,NOLEG	STAB1240
50	ANG(I)=2000.	STAB1250
	ANX=GSINZT*YSD	STAB1260
	ANY=-(GCOSZT*ZSD+GSINZT*XSD)	STAB1270
	ANZ=GCOSZT*YSD	STAB1280
	AMAG=SQRT(ANX**2+ANY**2+ANZ**2)	STAB1290
	ANX=ANX/AMAG	STAB1300
	ANY=ANY/AMAG	STAB1310
	ANZ=ANZ/AMAG	STAB1320
C		STAB1330
C	DETERMINE FOOTPAD POSITIONS RELEATIVE TO PLANE OF MOTION	STAB1340
C		STAB1350
	DO 100 I=1,NOLEG	STAB1360
	DX(I)=XFPS(1,I)-XS	STAB1370
	DY(I)=YFPS(1,I)-YS	STAB1380
	DZ(I)=ZFPS(1,I)-ZS	STAB1390
	DDOTN=ANX*DX(I)+ANY*DY(I)+ANZ*DZ(I)	STAB1400
	SGN(I)=SIGN(1.,DDOTN)	STAB1410
	IF(ABS(DDOTN).LE.1.E-05)SGN(I)=0.	STAB1420
100	CONTINUE	STAB1430
C		STAB1440
C	DETERMINE STABILITY ANGLE	STAB1450
C		STAB1460
	DO 200 I=1,NOLEG	STAB1470
	IF(SGN(I).NE.0.)GO TO 110	STAB1480
	XL=DX(I)	STAB1490
	YL=DY(I)	STAB1500
	ZL=DZ(I)	STAB1510
	GO TO 120	STAB1520
110	I1=I+1	STAB1530
	IF(I.EQ.NOLEG)I1=1	STAB1540
	ASGN=SGN(I)*SGN(I1)	STAB1550
	IF(ASGN.NE.-1.0)GO TO 200	STAB1560
	ANUM=ANX*DX(I)+ANY*DY(I)+ANZ*DZ(I)	STAB1570
	ADEN=ANZ*(DX(I1)-DX(I))+ANY*(DY(I1)-DY(I))+ANZ*(DZ(I1)-DZ(I))	STAB1580
	ALAMB=-ANUM/ADEN	STAB1590
	XL=DX(I)+ALAMB*(DX(I1)-DX(I))	STAB1600
	YL=DY(I)+ALAMB*(DY(I1)-DY(I))	STAB1610
	ZL=DZ(I)+ALAMB*(DZ(I1)-DZ(I))	STAB1620
120	CONTINUE	STAB1630
	GDOTL=-GCOSZT*XL+GSINZT*ZL	STAB1640
	ANG(I)=ACOS(GDOTL/(GRAV*SQRT(XL**2+YL**2+ZL**2)))	STAB1650
	GCLDN=GSINZT*YL*ANX-(GCOSZT*ZL+GSINZT*XL)*ANY+GCOSZT*YL*ANZ	STAB1660
	ANG(I)=ANG(I)*SIGN(1.,GCLDN)	STAB1670

200	CONTINUE	STAB1680
C		STAB1690
C	DETERMINE LANDER STABILITY	STAB1700
C		STAB1710
	KOUNT=0	STAB1720
	DO 300 I=1,NOLEG	STAB1730
	IF(ANG(I).EQ.2000.) GO TO 301	STAB1740
	IF(ANG(I).GT.0.)GO TO 320	STAB1750
	GO TO 300	STAB1760
301	KOUNT=KOUNT+1	STAB1770
300	CONTINUE	STAB1780
	IF(KOUNT .EQ.NOLEG)GO TO 310	STAB1790
C	PITCH INSTABILITY	STAB1800
	ISTAB=1	STAB1810
	RETURN	STAB1820
C	YAW INSTABILITY	STAB1830
310	CONTINUE	STAB1840
	ISTAB=2	STAB1850
	RETURN	STAB1860
C	LANDER STABLE	STAB1870
320	CONTINUE	STAB1880
	STAB=ANG(I)*57.295779513	STAB1890
	ROTX=DC(1,1)*WX+DC(1,2)*WY+DC(1,3)*WZ	STAB1900
	ROTY=DC(2,1)*WX+DC(2,2)*WY+DC(2,3)*WZ	STAB1910
	ROTZ=DC(3,1)*WX+DC(3,2)*WY+DC(3,3)*WZ	STAB1920
	STABVL=(ROTX*ANX+ROTY*ANY+ROTZ*ANZ)*57.295779513	STAB1930
	STABVL=SIGN(1.,(STAB-PRSTAB))*ABS(STABVL)	STAB1940
	IF(TIME.EQ.0.0)STABVL=0.0	STAB1950
	PRSTAB=STAB	STAB1960
	RETURN	STAB1970
	END	STAB1980

	SUBROUTINE MATINV (A, B, N)	MATV 10
C		MATV 20
C	COMPUTE THE SIMPLE MATRIX INVERSE OF A AND STORE IT IN B	MATV 30
C	A IS NOT ALTERED	MATV 40
C	A MUST BE NON-SINGULAR	MATV 50
C	A SHOULD HAVE DOMINANT DIAGONAL ELEMENTS AND BE OF	MATV 60
C	SMALL ORDER. IF THESE CONDITIONS ARE NOT MET,	MATV 70
C	THEN A DIFFERENT INVERSE ROUTINE SHOULD BE USED.	MATV 80
C	A = MATRIX OF ORDER (N,N)	MATV 90
C	B = INVERSE OF ORDER (N,N)	MATV 100
C	DIMENSION A(N,1), B(N,1)	MATV 110
C	COPY A TO B	MATV 120
	DO 100 I = 1, N	MATV 130
	DO 100 J = 1, N	MATV 140
	100 B(I,J) = A(I,J)	MATV 150
C		MATV 160
C	CONVERT B TO INVERSE BY G.E. ON BI WHERE I IS OVER B	MATV 170
C		MATV 180
C	IP = THE COL. BEING REDUCED TO ZEROS	MATV 190
	DO 400 IP = 1, N	MATV 200
	H = B(IP,IP)	MATV 210
C	FROM HERE ON CONSIDER THE IP COL. OF B TO BE THE IP COL. OF I	MATV 220
	B(IP,IP) = 1.	MATV 230
C	REDUCE DIAGONAL ELEMENT OF B TO 1, UPDATE I	MATV 240
	DO 200 J = 1, N	MATV 250
	200 B(IP,J) = B(IP,J) / H	MATV 260
C	REDUCE THE IP COL. OF B TO ZERO	MATV 270
	DO 400 J = 1, N	MATV 280
	IF (J .EQ. IP)GO TO 400	MATV 290
C	UPDATE IP COL. OF I	MATV 300
	H = B(J,IP)	MATV 310
	B(J,IP) = 0.0	MATV 320
	DO 300 K = 1, N	MATV 330
	B(J,K) = B(J,K) - B(IP,K)*H	MATV 340
	300 CONTINUE	MATV 350
	400 CONTINUE	MATV 360
	RETURN	MATV 370
	END	MATV 380

	SUBROUTINE GMPRD(A,B,R,N,M,L)	GMPD 10
C		GMPD 20
C	THIS SUBROUTINE MULTIPLIES TWO GENERAL MATRICES TO FORM A GENERAL	GMPD 30
C	RESULTANT MATRIX	GMPD 40
C		GMPD 50
	DIMENSION A(1),B(1),R(1)	GMPD 60
C		GMPD 70
	IR=0	GMPD 80
	IK=-M	GMPD 90
	DO 10 K=1,L	GMPD 100
	IK=IK+M	GMPD 110
	DO 10 J=1,N	GMPD 120
	IR=IR+1	GMPD 130
	JI=J-N	GMPD 140
	IB=IK	GMPD 150
	R(IR)=0	GMPD 160
	DO 10 I=1,M	GMPD 170
	JI=JI+N	GMPD 180
	IB=IB+1	GMPD 190
10	R(IR)=R(IR)+A(JI)*B(IB)	GMPD 200
	RETURN	GMPD 210
	END	GMPD 220

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