## Princeton University



## Department of <br> Aerospace and Mechanical Sciences


OPTIMAL CONTROL THEORY (OWEM)
APPLIED TO A HELICOPTER IN THE HOVER AND APPROACH PHASE By Gerard J. Born and Tadao Kaı
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#### Abstract

A major difficulty in the practical application of linear-quadratic regulator theory is how to choose the weighting matrices in quadratic cost functions. In this work, the control system design with optimal weighting matrices is applied to a hellcopter in the hover and approach phase. The welghting matrices are calculated to extremize the closed loop "Total System Damping" subject to constraints on the determinants. The extremization is really a minimization of the effects of disturbances, and intexpreted as a compromise between the generalized system accuracy and the generalized system response speed. The trade-off between the accuracy and the response speed is adjusted by a single parameter--the ratio of determinants, which is the only arbitrariness left for designer's choice.

By this approach an objective measure can be obtained for the design of a control system. The measure is to be determined by the system requirements.


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It is hoped the results of the research pursued under this program will provide the basis for further development by the Ames Research Center.

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```
a : velocity damping, sec
aij : (i,j) element of A
b : control derivative
c : noise derivative
c : (i,j) element of C
d : cutoff frequency of p.s.d. of gust model
e : error between command input and output
g(s) : transfer function
kij : (i,j) element of K
K
K}\mp@subsup{\dot{x}}{\quad: feedback gains of }{x}\mathrm{ and }\dot{x
pi
qi1 : (i-i) element of Q
r : command input or a scalar control weighting
s : Laplace operator s = \sigma # j\omega
s
s}
si
s}\mp@subsup{\mp@code{i+1}}{}{:\quad\mathrm{ eigenvalues of the autopiloted hovering helicopter}
u : m control vector
x : position state of a second order system
    : n state vector
y : p output vector
x(0) : initial position error
```

```
LIST OF SYMBOLS (CON'T.)
```

| A | : | nxn system matrix |
| :---: | :---: | :---: |
| $\mathrm{A}^{*}$ | : | $n \times n$ desirable decoupled system matrix |
| B | : | nxm control derivative matrix |
| B* | : | nxm desirable decoupled control derivative matrix |
| C | : | mxm control decoupling matrix, nxl noise derivatives |
| G | : | mxn feedback gain matrix |
| F | : | nxn closed loop system matrix |
| $G(J \omega)$ | : | frequency response of transfer function |
| I | : | identity matrix |
| $I_{i}$ | : | i-th order integral |
| ISE | : | integral squared error |
| ISU | : | integral squared control |
| IS $\theta$ | : | integral squared pitch attitude angle, $\mathrm{deg}^{2}$ sec. |
| IS $\dot{\phi}$ | : | integral squared roll attitude angle, deg ${ }^{2} \mathrm{sec}$. |
| IS $\psi$ | : | integral squared yawing angle, deg ${ }^{2} \mathrm{sec}$. |
| J | : | performance index cost |
| $J_{\text {min }}$ | : | minimum value of J |
| $K$ | : | nxn state feedback gain matrix for decoupling |
| $\overline{\mathrm{K}}$ | : | $n \times n$ decoupling matrix with defined eigenvalues |
| L | : | Lagrangian equation |
| M | : | $\mathrm{P}-\mathrm{Pe}$ |
| $M_{1}$ | : | 1 xl matrix |

## LIST OF SYMBOLS (CON'T.)



## LIST OF SYMBOLS (CON'T.)

Helicopter and V/STOL Notations
$x, y, z \quad: \quad$ longitudinal, lateral and vertical positions perturbations, ft.
$u, v, w \quad: \quad$ longitudinal, lateral and vertical velocity perturbations, ft/sec.
$\theta, \phi, \psi \quad: \quad$ pitch, roll and yaw attitude angle perturbations rad.
$\mathrm{p}, \mathrm{q}, \mathrm{r} \quad: \quad$ rolling, pitching and yawing angular velocity perturbations, rad/sec.

Uo, V : alrspeed, ft/sec
$\gamma_{0} \quad: \quad$ flight path angle, rad.
$X_{B 1 s} \quad: \quad$ longitudinal control input, rad.
A $\quad: \quad$ lateral cyclic control input, rad.
$\delta_{c} \quad: \quad$ collective control input, rad.
$\delta_{r} \quad: \quad$ yaw rudder control input, rad.
m : mass of helicopter, V/STOL, slugs
Ix,Iy,Iz : inertia moments about longıtudinal, lateral and vertical axis, slug $\mathrm{ft}^{2}$

Ixz : product of inertia moment, slug $\mathrm{ft}^{2}$
$X \quad: \quad$ longitudinal force, pound
$\mathrm{Xu}_{\mathrm{u}}, \mathrm{Xv}_{\mathrm{v}}, \mathrm{X}_{\mathrm{w}}$ : longitudinal force derivatives, $\frac{1 \partial \mathrm{X}}{\mathrm{m} \partial \mathrm{u}}, \frac{1 \partial \mathrm{X}}{\mathrm{m} \partial \mathrm{v}}$, etc. $\mathrm{Xp}, \mathrm{Xq}, \mathrm{Xr}$

Y : lateral force, pound

| $\begin{aligned} & \mathrm{Yu}, \mathrm{Y}_{\mathrm{V}}, \mathrm{Y}_{\mathrm{W}} \\ & \mathrm{Yp}, \mathrm{Yq}_{\mathrm{q}}, \mathrm{Yr}^{2} \end{aligned}$ | : lateral translational force derivatives, $\frac{1}{m} \frac{\partial Y}{\partial u}, \frac{1}{m} \frac{\partial Y}{\partial v}, \frac{1}{m} \frac{\partial Y}{\partial p}$, etc. |
| :---: | :---: |
| Z | vertical force, pound |
| $\mathrm{Zu}, \mathrm{Zv}, \mathrm{Zw}$ $\mathrm{Zp}, \mathrm{Zq}, \mathrm{Z}$ | : vertical translational force derivatives, $\frac{1}{m} \frac{\partial Z}{\partial u}, \frac{1}{m} \frac{\partial Z}{\partial v}, \frac{I}{m} \frac{\partial Z}{\partial w}$, etc. |
| L | rolling moment, ft. pound |
| $\begin{aligned} & \mathrm{Lu}, \mathrm{Lv}, \mathrm{Lw} \\ & \mathrm{Lp}, \mathrm{Lq}, \mathrm{Lr} \end{aligned}$ | $\begin{aligned} & : \text { rolling moment derivatives, } \frac{1}{I_{x}} \frac{\partial L}{\partial u}, \frac{1}{I_{x}} \frac{\partial L}{\partial v} \\ & \frac{1}{I_{x}} \frac{\partial L}{\partial w} \text {, etc. } \end{aligned}$ |
| M | : pitching moment, ft. pound |
| $\mathrm{Mu}, \mathrm{Mv}, \mathrm{Mw}$ $\mathrm{Mp}, \mathrm{Mq}, \mathrm{Mr}$ | : Pitchìng moment derıvative, $\frac{1}{I_{y}} \frac{\partial L}{\partial u}, \frac{1}{I_{y}} \frac{\partial L}{\partial v}$, $\frac{1}{I_{y}} \frac{\partial L}{\partial w}$, etc. |
| N | : yawing moment, ft. pound |
| $\mathrm{Nu}, \mathrm{Nv}, \mathrm{Nw}$ <br> $\mathrm{Np}, \mathrm{Nq}, \mathrm{Nr}$ | $\begin{aligned} & : \quad \text { Yawing moment derivatives, } \frac{1}{I_{z}} \frac{\partial N}{\partial u}, \frac{1}{I_{z}} \frac{\partial N}{\partial v}, \\ & \frac{1}{I_{z}} \frac{\partial N}{\partial \eta}, \text { etc. } \end{aligned}$ |
| $\mathrm{X}_{\text {B1s }}, \mathrm{X}_{\text {Als }}$ | : longitudinal force control sensitivity derivatives, |
| $\chi_{\delta_{c}}, x_{\delta_{r}}$ | $: \quad \frac{1}{m} \frac{\partial Y}{\partial \mathrm{~B}_{1 \mathrm{~s}}}, \frac{1}{\mathrm{~m}} \frac{\partial X}{\partial \mathrm{~A}_{1 s}}, \frac{1}{m} \frac{\partial X}{\partial \delta_{c}}, \quad \frac{1}{m} \frac{\partial X}{\partial \delta_{r}}$ |

## LIST OF SYMBOLS (CON'T.)

$Y_{\text {BIs }}, Y_{A l s}: \quad$ lateral force control sensitivity derivatives,
$Y_{\delta_{c}}, Y_{\delta_{r}} \quad: \quad \frac{1}{\mathrm{~m}} \frac{\partial Y}{B_{1 s}}, \frac{1}{m} \frac{\partial Y}{A_{1 s}}, \frac{1}{m} \frac{\partial Y}{\partial \delta_{c}}, \frac{1}{m} \frac{\partial Y}{\partial \delta_{r}}$
$Z, Z \quad: \quad$ vertical force control sensitivity derivatives,
$Z_{B 1 s}, Z_{A 1 s}: \frac{1}{m} \frac{\partial Z}{\partial B_{1 s}}, \frac{1}{m} \frac{\partial Z}{\partial A_{1 s}}, \frac{1}{m} \frac{\partial Z}{\partial \delta_{c}}, \frac{1}{m} \frac{\partial Z}{\partial \delta_{r}}$
$Z_{\delta_{c}}, Z_{\delta_{r}} \quad: \quad \frac{1}{m} \frac{\partial Z}{\partial B_{1 s}}, \frac{1}{m} \frac{\partial Z}{\partial A_{i s}}, \frac{1}{m} \frac{\partial Z}{\partial \delta_{c}}, \frac{1}{m} \frac{\partial Z}{\partial \delta_{r}}$
$L_{B 1 s}, L_{A l s}: \quad$ rolling moment control sensitivity derivatives,
$L_{\delta_{c}}, L_{\hat{\delta}_{r}}: \frac{1}{I_{x}} \frac{\partial L}{\partial B_{1 s}}, \frac{1}{I_{x}} \frac{\partial L}{\partial A_{1 s}}, \frac{1}{I_{x}} \frac{\partial L}{\partial \delta_{c}}, \frac{1}{I_{x}} \frac{\partial L}{\partial \delta_{r}}$
$M_{B 1 s}, M_{A 1 s}: \quad$ pitching moment control sensitivity
$M_{\delta_{c}}, M_{\delta_{r}} \quad: \quad$ derivatives, $\frac{1}{I_{y}} \frac{\partial M}{\partial B_{1 s}}, \frac{I}{I_{y}} \frac{\partial M}{\partial A_{I s}}, \frac{1}{I_{y}} \frac{\partial M}{\partial \delta_{c}}$,

$$
\frac{1}{I_{y}} \frac{\partial M}{\partial \delta_{r}}
$$

$\mathrm{N}_{\mathrm{Bls}}, \mathrm{N}_{\mathrm{Als}}$ : yawing moment control sensitivity derivatives,
$N_{\delta_{c}} N_{r} \quad: \quad \frac{1}{I_{z}} \frac{\partial N}{\partial \mathrm{~B}_{1 s}}, \frac{1}{I_{z}} \frac{\partial \mathrm{~N}}{\partial \mathrm{~A}_{1 \mathrm{~s}}}, \frac{1}{\mathrm{I}_{z}} \frac{\partial \mathrm{~N}}{\partial \delta_{c}}, \frac{1}{I_{z}} \frac{\partial \mathrm{~N}}{\partial \delta_{r}}$
$M_{\theta} n f, M c v: \quad$ pitching moment control sensitivity derivatives, $\frac{1}{m} \frac{\partial M}{\partial \delta_{n t}}, \frac{1}{m} \frac{\partial M}{\partial \delta_{v}}$

Xcv : longitudinal force control sensitivity derivatives,

$$
\frac{1}{m} \frac{\partial X}{\partial \delta_{v}}
$$

| Zcs, Znf | : vertical control force sensitivity derivatives, $\frac{1}{m} \frac{\partial Z}{\partial \delta_{s}}, \frac{I}{m} \frac{\partial Z}{\partial \delta_{n t}}$ |
| :---: | :---: |
| $\delta$ | : incremental collective fan input |
| $\delta_{n f}$ | : incremental nose fan input |
| $\delta_{s}$ | : incremental fan stagger input |
| [•] | new derivatives after cancelling inertia coupling terms |
| $[\cdot]$ | scaled computer variables |
| $K_{x}, K_{u}, K_{\theta}$, |  |
| $\mathrm{K}_{\mathrm{q}}$ | : (optimal) feedback gains of $x, u, \theta$, and $q$ |
| $K_{y}, K_{V}, K_{\phi}$, |  |
| $\mathrm{K}_{\mathrm{p}}$ | (optimal) feedback gains of $y, v, \phi$, and $p$ |
| $\alpha$ | any scalar number |
| $\delta$ | control input |
| $\delta_{o p}$ | optimal control input |
| $\delta(\cdot)$ | Dirac's delta function |
| $\triangle$ | : small variation, summation |
| $\rho^{2}$ | : trade off parameter |
| $\sigma$ | : rms value of random gust, $\mathrm{ft} / \mathrm{sec}$. |
| $\sigma_{x}, \sigma_{y}, \sigma_{z}$ | : rms longitudinal, lateral and vertical position errors, ft. |
| $\sigma_{\theta}, \delta_{\phi}, \delta_{\psi}$ | : rms pitch, roll and yaw attitude errors, degree. |
| $\lambda$ | : Lagrange's multiplıer |

## LIST OF SYMBOLS (CON'T.)

(1) : undamped natural frequency, rad/sec.
$\omega_{\mathrm{g}} \quad: \quad$ gust disturbance
$[\cdot]^{T} \quad: \quad$ transpose of a matrix
$[\cdot]^{-1} \quad: \quad$ inverse of a matrix
$|\cdot| \quad: \quad$ determinant of a matrix
cofactor
$\left(\mathrm{a}_{\mathrm{ij}}\right)$ : determinant of a matrix excepting i-th row and $j$-th column
$[\cdot]_{\mathrm{m}} \quad: \quad$ mean value
$\|\cdot\|:$ euclidean norm
[:] : time derivatıve
$T_{r}[\cdot] \quad: \quad$ summation of diagonal elements of a square matrix
$=, \cong \quad: \quad$ nearly equal to
$\mathcal{E}\{\cdot\} \quad: \quad$ expectation
$\{\cdot\}^{+}$: optimal

## CHAPTER I

INTRODUCTION
This study will explore the use of linear optimal control methods to design the automatic controller of a hovering helicopter. Generally the pilots workload is rather high for stabilizing a hovering helicopter [Fig. 2] and keeping position in the presence of gust over a designated point. Furthermore, to keep an accurate position may become more difficult if the motions [Fig. 3] among each axis are highly coupled.

In design of the multivariable and multi input systems, the conventional techniques such as root locus, frequency response, and describing function analysis which are based on the single input-single output systems, have been employed in trial and error fashion. Design techniques using the optimal control theory applied to multivariable systems in the state space have become popular and have been applied for the flight controller design or optımal control of aircraft [Ref. 1\&2].

The optimal control design technıque utilizes the minimızation of a performance criteria; one of which is a quadratic cost function, such as Eq. (3:1). Adequate selection of weightings in the quadratic cost function yzelds a stable closed loop system [Ref. 3]. However, the following questions remain to be answered:

How should the weighting matrices be.selected.
Where are the poles of the closed loop system in s plane located when control technique is used.

What are the system responses for a command input.
How is the system influenced by random gust inputs.

This work attempts to provide an answer to these questions and examines two methods for selecting weighting matrices, namely; the method of authoritıes [Ref. 6] and the optımal weighting selection method (OWEM) [Ref. 7]. The approach used in this investigation is as follows: First, the equation of motion of a helicopter comparable to a CH 54 B are decoupled, and the results are confirmed by an analogue computer study. Secondly, for each set of decoupled equations are derıved the optimal controls with its feedback gains. For a range of selected factor weightings, analysis has been made for the root loci of the closed loop system, optimal feedback gains, the integral squared position error (ISE), attıtude errors (IS日, IS $\phi$ ) and controls (ISU) to a command input. Also are investigated the root mean squared position errors, attitude errors and control motion for a random gust input. For the chosen pole location in $s$ plane, the closed loop gains are determined, and command inputs response (ISE, IS $\theta$, etc.) and gust responses (rms errors) are calculated.

In Chapter II, is reviewed the application of optimal linear control theory, and an outline of the performed work is gaven. Also is discussed the areas that need further study and investigation. ...........................

In Chapter III, the details of optimal controller design are described for the vertical and yawing motion, followed by the longitudinal and lateral motions of the hover helicopter.

In Chapter IV, conclusionsand recommendationson the optimal flight controller design using these methods are discussed.


#### Abstract

In the appendices are reviewed some of the theoretical aspects of linear optimal control theory and model following. It appears that the OWEM design method is also applicable to model following, as outlined in Chapter II. Theoretical work has been extracted from Ref. [7], applications and calculations from Ref. [34] and the Appendix [6] from Ref. [35].


## CHAPTER 2

REVIEN OF THE APPLICATION OF LINEAR OPTIMAL CONTROL THEORY

The purpose of this study is to apply optimal control methods for a controller design for helicopter at hover and in the approach phase. The design procedures in this report utilize first the decoupling of the helicopter dynamics (Appendix B). The system to be controlled consists of the decoupled helicopter dynamic equations.

In state variable form, the equations of motion are given by:

$$
\begin{equation*}
\dot{x}(t)=A x(t)+B u(t) \tag{2-1}
\end{equation*}
$$

In most cases, $x$ (n dimensional state vector), represents the state variables of motion of a hovering helicopter, which are, respectively, $x$, $u, \theta, q, z, w, y, v, \phi, p, \psi$, and $r$. The control variable $u$ (m dimensional vector), represents control inputs which are respectively $\mathrm{B}_{1 s}, \mathrm{~A}_{1 s}, \delta_{r}$, and $\delta_{c}$. In a few cases, the state vector x also can include the output of servo controller and so on.

The output vector $y(t)$ of helicopter is assumed to be the state vector $x(t)$ itself,

$$
\begin{align*}
& y(t)=H x(t)=x(t)  \tag{2-2}\\
& H=I: n x n \text { identıty matrix }
\end{align*}
$$

The advantages of state variable formulation is significant in the case of multi-variable and multi-input systems.

The design method uses an optimal control method with a quadratic performance index that minimizes the performance index given by the following equation:

$$
\begin{equation*}
J=\frac{1}{2} \int_{0}^{\infty}\left(x^{T}(t) Q x(t)+u^{T}(t) R u(t)\right) d t \tag{2-3}
\end{equation*}
$$

The matrices $Q$ and $R$ in Eq. (2-3) are respectively $n$ by $n$ and $n$ by m weighting matrices which place relative weights on quadratıc quantities $x^{2}, u^{2}, \theta^{2}, \ldots, B_{I s}{ }^{2} \ldots, \theta_{o}{ }^{2}$. These state variables represent small displacements and their incremental rates from an equilibrium point. Therefore, the minimization of the performance index (2-3) minimizes the energy in the state and control displacements.

The linear optimal quadratic control law yields linear feedback of states and results in a reasonable damped,stable closed loop system. The quadratic cost function may be regarded as a generalized weighted mean square error criterion for the multi-variable system working under the penalties on the control variables.

The quadratic cost function for this problem is written with a weighting matrix $Q$ for the state (or the output), and a scalar weighting $R=\rho^{2}$ for the control as follows:

$$
\begin{equation*}
J *=\operatorname{Min} \int_{0}^{\infty}\left[x^{T}(t) Q x(t)+\rho^{2} u^{2}(t)\right] d t \tag{2-4}
\end{equation*}
$$

The optimal control law $u(t)$ is obtaned as

$$
\begin{equation*}
u(t)=-R^{-1} B^{T} K x(t) \tag{2-5}
\end{equation*}
$$

It has been shown that the gain matrix $K$ must satisfy the algebraic matrix Riccati equation:

$$
\begin{equation*}
K A+A^{T} K-\frac{1}{\rho^{2}} K B B^{T} K+Q=0 \text {; where } Q>0 \tag{2-6}
\end{equation*}
$$

which yields a feedback control system with the stable, closed loop roots:

$$
R_{e}\left\{\lambda_{i}[F]\right\} \equiv R_{e}\left\{\lambda_{i}\left[A-\frac{1}{\rho^{2}} B B^{T} K\right]\right\}<0 \text { for all } i=1,2,3,4
$$

To solve Eq. $(2-6)$ for $K$ matrix, the weighting matrix $Q$ must be selected. For example, the longitudinal dynamic equations for a hovering helicopter (4th order system), one can write:

$$
\begin{equation*}
u(t)=-R^{-1} B^{T} K x(t)=-G_{1} x_{1}(t)-G_{2} x_{2}(t)-G_{3} x_{3}(t)-G_{4} x_{4}(t) \tag{2-7}
\end{equation*}
$$

where

$$
\begin{aligned}
& K \equiv\left\{k_{i j}\right\} ; i, j=1,2,3,4 \\
& G_{1}=\left(X_{B 1} k_{21}+M_{B 1} k_{41}\right) / \rho^{2} \\
& G_{2}=\left(X_{B 1} k_{22}+M_{B 1} k_{42}\right) / \rho^{2} \\
& G_{3}=\left(X_{B 1} k_{23}+M_{B I} k_{43}\right) / \rho^{2} \\
& G_{4}=\left(X_{B 1} k_{24}+M_{B I} k_{44}\right) / \rho^{2}
\end{aligned}
$$

After the Laplace transform, the control law Eq. (2-7) can be written as:

$$
\begin{equation*}
B_{1 s}(s)=-G_{x}\left(T_{x} s+1\right) x(s)-G_{\theta}\left(T_{\theta} s+1\right) \theta(s) \tag{2-8}
\end{equation*}
$$

where

$$
\begin{equation*}
G_{x}=G_{1}, G=G_{3}, T_{x}=G_{2} / G_{1}, T_{\theta}=G_{4} / G_{3} \tag{2-9}
\end{equation*}
$$

The optimal control system theory attempts to provide an analytical design procedure that lessens the designer's load in the design task, and locates more of the load on the computational machines. The problem $1 s$ to find an optamal linear controller for a linear plant by minimizing the above quadratic system performance criterion or a quadratic cost function. The optimal linear-quadratic control theory provides a well organized design procedure for a linear system with a feedback structure. The linear feedback structure would probably be, from the practical point of view, the most beneficial result of optimal linear-quadratic control theory. The application of the theory, even the multiloop system design, which is often difficult in the conventional approach, may be thought to become easıly accessible for system designers. All the possible feedback loops are taken into account and all the corresponding feedback gains are determined in the design procedure. The resulting closed loop system appears to be not only a stable system, but also a minimum weighted mean square error system.

The benefits and the mathematical beauty of the optimal linearquadratic control theory mi-ght give a farse impression that almost all the difficulties in linear system design have been elıminated. The entire theory has been developed under the presumption that the welghting matrices in the quadratic cost function are somehow given a priorı. They are completely left open for the designer's choice.

The wexghting matrices are usually selected only for their diagonal elements by the practical system designer on the basis of his engineering experience, coupled with simulation runs for different trial values. One can be obliged to do a tediously long trial-and-error approach, especially in the design of multiloop systems., Many of the attractive features of optimal linear-quadratic control theory seem to disappear. The optimal linear-quadratic control theory may be even worse than the conventional methods for multiloop system designs, because the weaghting matrices have no apparent quantitative relationships with the closed loop system dynamical characteristics. None the less, the welghting matrices determine, to a large extent, on the resulting system dynamical characteristics. Hence, the selection of weighting matrices often become a considerably difficult task. One may be able to say that most of the potential difficulties of multiloop system design have been concentrated on the selection of weighting matrices, and the optimal control theory does not ease essential difficulties in the system design.

Accordingly, the selection of weighting matrices really has a crucial significance in the practical system design through the optimal linearquadratic control theory. Some of the qualitative properties of weighting
matrices are summarized in Ref. [26]. They may help designers to select the trial weighting matrices. In order for the theory to be practically applıcable, however, it is necessary to develop a reasonable and unified way for the quantitative selection of weighting matrices. This is one of the topics considered in this work.

Two methods for the quantitative selection of these weighting matrices have been avallable.

1. Method of Authorities of Varıables (Refs. [27] and [28])

This method is based on the normalization of output variables and control variables. Every weighting factor for the output variables and the control variables is inversely proportional to the square of the prescribed maximum for that variable. The quadratic cost function becomes a linear combination of normalized mean square errors and controls. This method only provides an initial estimate for the weighting matrices, which are assumed to be of diagonal forms. However, there are no clear-cut reasons for the weighting matrices to be necessarily of a diagonal form, since the output components are usually coupled with one another. In order to refine the resulting system, a trial-and-error approach would be inevitable. Besides, it is often a difficult task to estimate the maximum values of all the output and the control components. Even if it is possible, there is no guarantee that all the outputs and the controls of the resulting system are suppressed within the prescribed maximum values.
2. Closed Loop Pole Allocation (Refs. [20] and [6])

This method achieves the specified closed loop poles by appropriately
choosing the weighting matrices in the quadratic cost function. The original work for this method has probābly been done by E.J. Ellert (Ref. [6 ]).
 assigned poles on the complex plane with some linear feedback control law, 1f the open loop system is completely controllable. However, this has been proved independently of quadratic cost functions. More recent studies (Refs. [36] and [37]) on the Inverse problem of linear feedback control show that every linear feedback control minimizes a quadratic cost function with certain weighting matrices. Combining Wonham's work and the results out of the inverse problem, it is possible to place the closed loop poles anywhere on the complex plane by choosing appropriate weighting matrices. However, the difficulty is how to allocate all (but not some) of the closed loop poles, especially in high-order systems. The domant closed loop pole allocation has also been studied (Ref.[20]). This is a method to approximately obtan the desired dominant poles. However, this does not answer the essential difficulties in deciding the number of dominant poles, and in choosing the domınant poles. Besides, this approach usually yıelds a system with very high feedback gains, and requires complicated computational procedures. If the desired closed loop poles can be assigned, the optimal linear-quadratic control theory may not be necessary, but some other direct approach may be more convenient (example: Ref. [31]).

Thus, there is still a large gap between the optımal linear-quadratic control theory and its application to practical system design. Until such a gap is bridged by developing a reasonable, simple, and well-unified method to select the weighting matrices, the optimal linear-quadratic control theory would reman to be just a "theory" and would not be accessible to the
designers of practical systems.
This work is based upon Ref. [7], which attempts to bridge the gap. Below and in Appendix $E$ is given a summary of the two new methods described in Ref. [7], to select the weighting matrices of general forms not necessarily diagonal matrices).
3. OWEM Design Method (Ref. [7])

Based on the thought that the quadratic cost function is a legitimate system performance measure in the linear-quadratic control theory, the minimum quadratic cost function with the optımal linear feedback control law is actively used as the criterion to select the weighting matrices. First, the nature of the minimum quadratic cost function has been examined for particular sequences of weighting matrices. It leads to the necessity of some constraints on the weighting matrices, as a minimum equal to zero occurs when the weightang matrices are zero. In Ref. [7], reasonable constraints are explored to make the optimal weighting matrices not equal to zero or finite, and this appears to be a determinant constraint. Under this constraint, the minimum quadratic cost function is further minimized with respect to weighting matrices. The resulting weighting matrices are based upon the system noise characteristics, and hence, the minimum of the quadratic cost function becomes a function of the noise $N$ or $J_{\min }(N, Q, R)$. The resulting closed loop system dynamics is a system that minimizes the effect of the noise or disturbance for which it was designed. It was also shown that the resulting optimal weighting matrices $Q$ and $R$ were inversely proportional to the output covariance matrix, and the induced covariance matrix of optimal controls, respectively. Obviously, they are not necessarily diagonal matrices. Note that these results are quite similar to the weightings proposed by Gauss for the weighted least-square estimation problem.

The optimal weighting matrices appropriately normalize the output and the control variables, and also equalize in groupwise all the additive terms appearing in the quadratic cost function. The two functions, i.e., the normalization and the equalization, are fundamental roles required for the welghting matrices in quadratic cost functions. Because of the normalizing effect of weighting matrices, the resulting closed loop system dynamacs became irrelevant to the physical dimensions which were used to express the plant dynamical equations.

It was also shown that the ratio $\rho=|R| /|Q|$ between the prescribed determinants of $R$ and $Q$ represented some kind of measure about the tradeoff between the contribution of control variables, and that of system output variables to the quadratic cost function. The tradeoff parameter $\rho$ would play the key role for determining the closed loop system bandwidth.

The method described above provided a well formulated set of necessary and sufficient conditions for reasonable optimal weighting matrices with only one unknown parameter or trade-off parameter equal to $\rho$. It required the a priori knowledge about the system noise intensity matrix $N$ since the criterion $J_{\text {min }}(N ; Q, R)$ was a function of $N$. The a priori knowledge about the plant noise may not be available in many cases. Then, the second method for the selection of the weighting matrices was developed by introducing a new criterion which is independent of the noise intensity matrix $N$.

The concept of n-dimensional hyper error ellipsoid (state error set) is used and a measure of the state error set was defined by its n-dimensional volume $(V(t))$. The "Relative Rate of Convergence of Error Sets" was defined as $\left|\frac{1}{V(t)} \frac{d V(t)}{d t}\right|$, and this represents a sort of generalized system response speed. This new criterion is independent of the plant noise intensity
matrix $N$, and was shown to be exactly the same as the so-called "Total System Damping". The well formulated necessary and sufficient conditions for the noise-free optimal weighting matrices are derived as the solution of the Mini-Max problem for the RERACES (or TSD).

It has been shown that, using the second method selection of the $Q$ matrix yields the same result as the first method, if the noise characteristics (see Fig. 1) are assumed to be white noise disturbances with $N_{e}=k R^{-1}$. In other words, a white noise disturbance in $u(t)$. The effect of noise or disturbance bandwidths in the helicopter application is discussed in Chapter 3 , and is often small for the range of operating conditions used in that application.

The initial application of this method for determining the Optimal Welghting Matrices to second through fifth order example in Ref. [38] indicates the advantages of this approach. For these examples, the conventional and optimal methods have been used extensively, and the results are well known. For these examples, the area of the pole location for the precision hover of a helicopter has been well established, and each of the above methods can be applied yielding (often by trial and error), about the same results. However, it has been demonstrated, by these examples, that the application of the OWEM method yields, without a trial and exror method, a good and well damped solution to the problem.

This solution is independent of scaling (dimensions) of the problem statement, and assumes in the first instance, that there is no a priori different penalty on error, gain,frequency, etc., among the state variables.

In summary, for the OWEM design method it is required to know:

1. the system matrix
2. the control matrix
3. the noise or disturbance matrix

The OWEM design method then determines the weighting matrices $Q$ and $R$ for minimizing the quadratic performance index under the assumption that all errors are equally important, and hence, these matrices for the calculation of the feedback gains. When the noise $N$ is unknown, Q and R can be computed by the mini-maximization of TSD with respect to $Q$ 'and $R$ under prescribed determinant constraints. This is equivalent to a white noise disturbance with a magnitude $N_{e}$ inversely proportional to the control weighting $R$, and the minimization of TSD is equivalent to the minimization of $J_{\text {min }}\left(N_{e} ; Q, R\right)$.

The OWEM design method is based upon minimization of known or estimated disturbances and results in a control system with adequate stability or damping. In the generalized formulation no consideration is given to system limitations, such as acceleration llmits, gain limitatıons, and so on. Hence, in the design of a flight controller, the performance and control boundaries must be considered. These boundaries are essential in the practical application of the design method. For example, take the case of a hovering helicopter (longitudinal).

A design is to be made for an automatic control system. However, the assumption is made that when the system is switched to manual control, it is desirable that the mode frequencies do not change appreciably. Hence,
the boundaries for the automatic system should be similar to those of manual control. The longitudinal control of the precision hover of a helicopter is in principle dominated by 2 modes; a "position" mode corresponding to the "outer loop" and a "attitude mode" corresponding to the "inner loop" of the feed control system.

An estimate of the boundaries of the "position" mode is given in Ref. [22], and summarized below.

A massless point controlled according to a linear error control law, $\dot{\mathrm{e}}=-\mathrm{Ke}$, would result in a first order system with a closed loop bandwidth $\omega_{C L}=1 / \tau_{C L}=K$. Instantaneous speed changes not being possible, even the simplest model has to incorporate the inertia. This leads to a second order system for control. The most significant assumption in this very simple model is that tilting the thrust vector and changang its magnitude, is possible with negligible delay compared with the position keeping bandwidth. The block diagram of such a simplified model is shown in Illustration (2-1 ).


Illustration 2-1.
Simplified Block Diagram of a Hovering Helicopter

The input to the long1tudinal positioning loop is shown as $\Delta a_{N}$, the acceleration disturbance. The gains $\bar{g} G_{x}$ and $g g_{\dot{x}}$ determine the acceleration per foot exror and per $f t / s e c$ error rate, respectively. The following fundamental relationships are easily deduced from the block diagram.

The natural frequency and therefore, the bandwidth of the position loop is

$$
\begin{equation*}
\omega_{\mathrm{p}}=\sqrt{\mathrm{g} G_{\mathrm{x}}} ; \tag{2-10}
\end{equation*}
$$

$g G_{x}$ is the "spring constant" of the position loop. The relative damping ratio is

$$
\begin{equation*}
\zeta_{p}=(\sqrt{g} / 2) \quad\left(G_{x}^{\cdot} / \sqrt{G_{x}}\right)=2.83\left(G_{x} / \sqrt{G_{x}}\right) \tag{2-11}
\end{equation*}
$$

The relationship between the bandwidth and the damping ratio is determined by

$$
\begin{equation*}
G_{x} / G_{x}=2\left(\zeta_{p} / \omega_{p}\right) \tag{2-12}
\end{equation*}
$$

and, for a predetermined bandwidth and damping ratio

$$
G_{x}=(I / g) \omega_{p}^{2} \text { and } G_{x}^{*}=(2 / g) \zeta_{p} \omega_{p}
$$

In the case of a constant acceleration disturbance, the steady state error is $a g / \omega_{p}^{2}=a / G_{x}$. The simple model provides reasonable estimates of the
relationship between bandwidth gains and errors as long as the bandwidth of the attitude control loops is several tames faster than that of the station keeping loop. Such a separation is feasible and appears desirable in most applications.

For the choice of bandw1dth of the position loop, the following considerations can be made. A large $\omega_{p}$ leads to a high error gain $G_{X}$, a small $\omega_{p}$ results in a large steady state error for constant acceleration disturbance.

The table below shows some characteristic values of bandwidth, gain and error.

| $\omega_{p}(\mathrm{rad} / \mathrm{sec})$ | .1 | .2 | .5 | 1.0 |
| :--- | :--- | :--- | :--- | :--- |
| $G_{X}\left(\mathrm{deg} / \mathrm{ft}^{2}\right)$ | .018 | .07 | .45 | 1.8 |

steady state error (ft)
for . 1 g acceleration 320 80 13
3.2

This table helps to narrow the choice of $\omega_{p}$ to between approximately .3 and .5. With values under . 3 , the error becomes excessive, above .5 the thrust angle variations with even small errors are rapidly becoming too large.

The desirability to make the bandwidth of the "attitude" mode faster than that of the "position" mode is limited by the manual attitude control loop. This results in an "attitude" mode of frequency in the order of $\omega>$ 1.0 to $2 \mathrm{rad} / \mathrm{sec}$.

In the conventional control system design method, the gain limitations are taken into account for choosing the "inner" and "outer" loops. In these
hovering helicopter examples a "tight" inner loop (altitude) is designed with sufficient damping by adjusting $G_{\theta}$ and $G_{\dot{\theta}}$, the outer Ioop (position) is determined by the position gain 1 imitation $G_{X}$; the velocity gain $G_{\dot{X}}$ is adjusted to provide sufficient damping.

In the OWEM design method as developed does not take into consideration gain limitations. When there are no gain limitations, the operating point in the linear quadratic optimal design is determined by the control weighting. This number is determined by the control limitations, and the maximum expected disturbances, and/or command inputs.

However, the OWEM design method can be easily adapted by additional information, into the design.

When no additional information is used, the computational procedure regards the errors and the associated feedback gains, etc., equally important. Hence, in this design method, the root location tends to be circular with more damping than the Butterworth configuration. After the initial design is made, the resulting mode frequencies, errors, gains, etc., can be analyzed. When the value of these quantities are not in conflict with the practical design requirements, then one can consider the design to be completed.

However, in many cases, practical physical limitations have to be imposed on errors, loop gains, mode frequencies, etc. For example, in the hovering helicopter, the position mode frequency is acceleration limited e.g., . 3 to . 5 radians $/ \mathrm{sec}$.

On the other hand, the attitude loop should be a much higher frequency, e.g., $1-3 \mathrm{rad} / \mathrm{sec}$.

In Appendrx $E$ is shown in the calculation of the optimal weaghting matrices the groups wise equalization matrix (S) is equal to the unity T matrix. This imposes an equal weighting on the elements of the $y^{*} Q y^{*}$. In engineering terms, this implies that all the mean square error associated control, gains, etc., are of equal importance.

The initial design is done on the digital computer using $S=I$. The resulting design is then examined with respect to errors, control and gain limitations. For example, one can examine errors and mode frequencies as a function of the control weighting $\rho$. When this examination shows that the control weighting $\rho$ determined by the control limitation, no other boundaries (imposed or desired) are violated, then the design is finıshed. If, however, one finds that for the desired control weighting $\rho$, one of the predetermined requirements is not met or gain boundaries are violated, then one should determine the violated boundary on the Optimal Root Location (ORL) plot.

In the example shown for the longitudinal hovering helicopter, the position acceleration limitation limits the $\rho$ to approximately equal to 800 (. 3 to $.5 \mathrm{rad} / \mathrm{sec}$. ), while the desired altitude boundary yields a $\rho$ approximately equal to 3200 . This implies that in order to reach the control boundary and acceleration boundary at the same time, one should use an equalization matrix which has a value of $1 / 4$ for the state variables associated with this limitation.

In other words, one can assign a different weighting on the system errors if this also is equivalent to assigning a diagonal Error Scaling Matrix (ESM). The designer is free to investigate the effect of the ESM matrix upon the final system design. Note that the ESM matrix is a diagonal matrix, so that the elenents of the matrix are easily interpreted.

In this manner, an objective measure of an error criteria is used to determine the weighting matrices. The approach taken in this work Imposes the boundaries more or less by engineering judgement in terms of errors. A very important further theoretical investıgation with significant practical aspects $1 s$ the introduction of constraints in the OWEM design.

The OWEM design method is also directly applicable in model following for the determination of the weighting matrix $\hat{Q}$ (Appendix G1, Eq. G1-22). The weighting matrix $\hat{Q}$ in the model reference system is composed of the original weighting factors $Q$ of 1 nnear feedback systems. The OWEM method determines the $Q$ matrix for minimization of disturbances. Hence, a we1ghting matrix determined by this principle should yield a prefilter model reference control system that minimazes the errors due to the disturbances.

Another important investigation will be the practical aspects of the use of OWEM design criteria for incomplete measurement and feedback of all state varıables.

## CHAPTER 3

## CONTROLLER DESIGN FOR A HELICOPTER (ONEM)

1 A Helicopter system and control equations
In this chapter the optimal control design will be illustrated on the controller design of a helicopter. The system to be controlled is a system representative of an uncoupled hoverıng helicopter CH54B given by Eq. 3-1.


For the optimal design, the following equations are used and solved with the aid of a digital computer.

$$
\begin{align*}
& \text { System dynamics: } \dot{x}(t)=A x(t)+B u(t)+C n(t)  \tag{3.2}\\
& y(t)=H x(t)  \tag{3.3}\\
& \text { where } x(t) \text { : } n \text { dimensional state vector } \\
& u(t) \text { : } m \text { dimensional control vector ( } m<n \text { ) } \\
& y(t) \text { : } p \text { dimensional output vector ( } p<\bar{n} \text { ) } \\
& n(t): \ell \text { dimensional noise vector ( } \ell \leq \bar{n} \text { ) } \\
& \mathcal{E}\{(\mathrm{n}) \mathrm{\}}=0 \\
& \operatorname{Cov}\{n(t) ; n(\tau)\}=N \delta(t-\tau)(\text { unknown }) \\
& A, B, C \text { and } H \text { : nxn, nxm, nxl and pxn constant } \\
& \text { matrices, respectively. }
\end{align*}
$$

Cost function: $J_{\min }(\mathbb{N} ; Q, R)=\underset{u(t)}{\operatorname{Min}}\left\{\begin{array}{l}\operatorname{Lim} \frac{\bar{T}}{\operatorname{Lim}} \\ \frac{\bar{T}}{T} \\ \int_{-T}^{T}\end{array}\left[y^{T}(t) Q y(t)+u^{T}(t) R u(t)\right] d t\right\}$

$$
\begin{equation*}
=\operatorname{Tr}\left[\operatorname{CNC}^{T} \mathrm{~K}\right] . \text { (N: unknown) } \tag{3.4}
\end{equation*}
$$

$$
\begin{align*}
& \text { If } N_{e}=k C^{+} B R^{-1} B^{T}\left(C^{+}\right)^{T} \text {, then } \\
& J_{\min }\left(\mathbb{N}_{e} ; Q, R\right)=k T r\left[B R^{-1} B^{T} K\right] \\
& \text { where } C^{+}: \text {\&xn pseudo-inverse matrix of } C \text {,i.e., } \\
& \quad\left(=\left(C^{T} C\right)^{-1} C^{T}\right) \tag{3.6}
\end{align*}
$$

Optimal control law: $u(t)=-R^{-1} B^{T} K x(t)$
where K: nxn unique positive definite solution of (3.7) $K A+A^{T} K-K B R^{-1} B^{T} K+H^{T} Q H=0$

The optimal control law (3.7) can be computed without knowing $N$ if $Q$ and $R$ are given.

If the $Q$ and $R$ matrices are estimated, then for these $Q$ and $R$ values the optimal control law can be calculated.

When the values for the matrices $Q$ and $R$ are not estimated, then one needs an additional criteria. For example, minimızing total system error in total system damping, etc. as discussed in Ref. (Z). Assuming that gust or noise information is not available, the selection of optimal weighting matrix is done by minamazing the criterion $L\left(Q, \rho^{2}\right)$, as discussed in Ref. 7 and Appendix E.

## Auxiliary performance index

Total Systen Damping $=$ TSD: the negative sum of the closed loop roots $L^{+}=\underset{Q}{\operatorname{Min}} \underset{R}{\operatorname{Max}}\{T S D\}=\underset{Q}{\operatorname{Min}} \underset{R}{\operatorname{Max}}\left\{\operatorname{Tr}\left[-\left(A-B R^{-1} B^{T} K\right)\right]\right\}$
or equivalently

$$
\begin{aligned}
& L^{*}=\operatorname{Min}_{Q}^{\operatorname{Max}}\left\{\operatorname{Tr}\left[B R^{-1} B^{T} K\right]\right\}(\because A=\text { constant matrix }) \\
& \text { subject to } K A+A^{T} K-K B R^{-1} B^{T} K+H^{T} Q H=0
\end{aligned}
$$

Note: (1) TSD is equivalent to the Relative Rate of Change of Error Sets (RERACES, see Ref. [7], Chapter IV), and is a generalized system response speed.
(2) The extremum (minimax) of TSD does not exist unless $Q$ and $R$ are appropriately constrained.

## Constraints on $Q$ and $R$

$$
|Q|=1,|R|=\rho^{2}
$$

where

$$
\begin{equation*}
|(\cdot)|: \text { the determinant of }(\cdot) \tag{3.9}
\end{equation*}
$$

## Optimal weighting matrices

$$
\begin{equation*}
Q=P \sqrt{\left|\mathrm{HPH}^{T}\right|} \cdot\left(\mathrm{HPH}^{T}\right)^{-1} \tag{3.10}
\end{equation*}
$$

$$
\begin{equation*}
R=\rho^{2 / m} \frac{B^{T} K B-B^{T} K P K B}{\sqrt[m]{\left|B^{T} K B-B^{T} K P K B\right|}} \tag{3.11}
\end{equation*}
$$

$$
\begin{gathered}
\text { where } F P+P F^{T}+B R^{-1} B^{T}=0 \\
F=A-B R^{-1} B^{T} K
\end{gathered}
$$

The latter equations are solved on the digital computer for the matrix Q. The optimal weighting matrices are then used in the Riccati Eq. 3-8 which yields the optimal control law (Eq. 3-7).

For a hovering helicopter CH54B, the numerical values for Eq. 3-2 are shown in Eq. 3-13.

$$
\begin{align*}
& \mathrm{A}=\left[\begin{array}{ccc:c:ccc:c}
-.0169 & -32.2 & .661 & 0.0 & 0.0 & 0 & 0.0 & 0.0 \\
& & 1.00 & & & & & \\
0.0024 & 0 & -.257 & 0.0 & 0.0 & 0 & 0.0 & 0.0 \\
\hdashline 0.0 & 0 & 0.0 & & 0.269 & 0.0 & 0 & 0.0 \\
\hdashline 0.0 & 0 & 0.0 & 0.0 & 0.0 \\
\hdashline & & & 0.0 & 0 & 32.2 & -.895 & 0.0 \\
0.0 & 0 & 0.0 & 0.0 & -.0200 & 0 & -1.14 & 0.0 \\
\hdashline 0.0 & 0 & 0.0 & 0.0 & 0.0 & 0 & 0.0 & -.591
\end{array}\right] \\
& B=\left[\begin{array}{ccccccc}
35.6 & 1 & 0.0 & 1 & 0.0 & 1 & 0.0 \\
0 & 1 & 0 & 1 & 0 & 1 & 0 \\
-5.66 & 0.0 & 0.0 & 0.0 \\
\hdashline 0.0 & -292 . & 0.0 & 0.0 \\
\hdashline-0 & 1 & 0.0 & 36.1 & 0.0 \\
0 & 1 & 0 & 1 & 0 & 1 & 0 \\
0.0 & 1 & 0.0 & 24.6 & 0 \\
0.0 & 1 & 0.0 & 0.0 & -10.9
\end{array}\right] \tag{3.13}
\end{align*}
$$

These equations can be divided into four separate systems of equatinns, namely:

1. Vertical Control System
2. Yaw Control System
3. Longitudinal Control System
4. Lateral Control System

For a helicopter in the approach phase, the numerical values for Eq. 3-2 are shown below in Eq. 3-14.

$$
A^{*}=\left[\begin{array}{ccc:c:ccc:c}
-.0221 & -32.2 & 1.515 & 0.0 & 0.0 & 0 & 0.0 & 0.0 \\
& & 1.0 & : & & & & \\
0 & & 0 & -.4251 & 0.0 & 0.0 & 0.0 & 0 \\
\hdashline 0.0 & 0 & 0.0 & 0.0 & 0.0 \\
\hdashline 0.0 & 0 & 0.0 & 0.0 & -.0711 & 32.2 & -1.788 & 0.0 \\
& & & 0 & & & & 0.0 \\
& & 0.0 & 0.0 & -.0113 & 0 & -1.488 & 0.0 \\
0.0 & 0 & 0.0 & 0.0 & 0.0 & 0 & 0.0 & -.937
\end{array}\right]
$$

## B Controller Design for Vertical and Yaw Control

In hovering, the helicopters have inherent rate damping in vertical and yawing motions with stable vertical speed and yaw rate characteristics. However, with regard to the position stability, both the position control equations are neutrally stable due to pole located at origin. This means that the helicopter, if uncontrolled and if exposed to disturbances, will drift away from desired hover points. Taking these into account, the following design objectives are recommended for precision hovering.
(1) To obtain the stable well damped position control system
(2) Good system response to position command input, with a small or no overshoot, and minimizing integral square error (ISE)
(3) Minimizing root mean square position errors and resulting controls to random gust input.

Some of these requirements oppose each other and a compromise becomes necessary.

The characteristics of the decoupled (uncoupled) motion of vertical and yawing equations are of the second order rate command systems as in Eq. (3-14).

$$
\begin{equation*}
\ddot{x}(t)+a \dot{x}(t)=b \delta(t)+\left(c w_{g}(t)\right) \tag{3-15}
\end{equation*}
$$

where
$x(t):$ position $Z$, ft or $\phi$, rad
$\dot{x}(t)$ : rate of position $Z, f t / s e c$ or $r, ~ r a d / s e c$
a. : rate damping $Z_{w}$ or $\mathrm{Nr}, \mathrm{sec}^{-1}$

```
            b : control derivatives }\mp@subsup{Z}{0}{}\mathrm{ , ft/ sec
    \delta(t): control input }\mp@subsup{\delta}{c}{}\mathrm{ or }\mp@subsup{\delta}{r}{},ra
wg
    c : gust derivatives, }\mp@subsup{Z}{w}{},1/\textrm{sec}\mathrm{ or N N
```

We assume for calculation purposes that disturbances even in decoupled yaw equation has effect on control through $N_{v}$ term. Design Method by Conventional Selection of Werghting Factors

The optimal control is derived by minimizing, in a disturbed system the following integral without $\mathrm{W}_{\mathrm{g}}$

$$
\begin{equation*}
\frac{1}{2} \quad \int_{0}^{\infty}\left(q_{11} x^{2}(t)+q_{22} \dot{x}^{2}(t)+r \delta^{2}(t)\right) d t \tag{3-16}
\end{equation*}
$$

$$
\begin{aligned}
\text { Where } q_{11} & : \text { weighting factor for position } \\
\mathrm{q}_{22} & \text { : weighting factor for rate } \\
r & : \text { weighting factor for control }
\end{aligned}
$$

Selection of Weighting Elements $q_{11}, q_{22}$ and $r$

In both systems the closed loop system characteristics are covered by selection of non-zero diagonal $q_{11}, q_{22}$ and $r$ only, as shown in Appendix $D$. The weighted quadratic terms in the integral should be non-dimensionalized. The following units are selected for $q_{11}, q_{22}$ and $r$.

Vertical
$\mathrm{q}_{11} \mathrm{ft}^{-2} \quad \mathrm{rad}^{-2}$ $q_{22} \quad \mathrm{ft}^{-2} \cdot \mathrm{sec}^{2} \quad \mathrm{rad}^{-2} \mathrm{sec}^{2}$
$r \quad \mathrm{rad}^{-2} \mathrm{rad}^{-2}$

The values of weighting factors are specified by choosing an equivalence of the values of position errors $(Z, \phi)$, rate errors $(\dot{Z}, r)$ and resulting control ( $\delta_{c}, \delta_{r}$ ). The selection is as follows:

$$
\begin{array}{ll}
\text { Vertical }(10 \mathrm{ft})^{2}=(5 \mathrm{ft} / \mathrm{sec})^{2}=(0.1 \mathrm{rad})^{2} \\
\text { Yawing } & (0.2 \mathrm{rad})^{2}=(0.1 \mathrm{rad} / \mathrm{sec})^{2}=(0.1 \mathrm{rad})^{2} \tag{3-19}
\end{array}
$$

These weighting factors correspond to the case (5) with $r=100$ in Table 1 below, where a number of computed numerical examples are listed.

Table 3.1. Computed Numerical Examples
Case No.
Weighting factors
$q_{11} \quad q_{22} \quad r$

$$
0 \rightarrow \infty
$$

1.0
0.25

$$
0 \rightarrow \infty
$$

1.0
1.0
$0 \rightarrow \infty$
1.0
$2.0 \quad 0 \rightarrow \infty$
1.0
4.0
$0 \rightarrow \infty$
1.0
16.0
$0 \rightarrow \infty$

Note that only the relative magnitudes (values) have effects on the characteristics of the closed loop system (see Appendix D).

Optimal Control. The optimal control to minimize the performance index (Eq. 3-16) is given as function of $q_{11}, q_{22}$ and $r$ (Appendix D).

$$
\begin{equation*}
\delta_{o p}(t)=-\sqrt{\frac{q_{11}}{r}} x-\left(\frac{a}{b}+\sqrt{\left(\frac{a}{b}\right)^{2}+\frac{2}{b} \sqrt{\frac{q_{11}}{r}}+\frac{q_{22}}{r}}\right) \dot{x}(t) \tag{3-20}
\end{equation*}
$$

Substituting this into Eq. (3-15) gives the closed loop system

$$
\begin{equation*}
\ddot{x}(t)+\sqrt{a^{2}+2 b \sqrt{\frac{q_{11}}{r}}+b^{2} \frac{q_{22}}{r}} \dot{x}(t)+b \sqrt{\frac{q_{11}}{r}} x(t)=0 \tag{3-21}
\end{equation*}
$$

This derivation and its properties are shown in Appendix D.
The optimal control, in general, has been solved for the system without noise as shown in Appendix D. The separation theory in Refs. $\hat{2} \hat{0}, 27,39$ ) warrants the optimal control to be optimal for the system with gaussian white noise. However, unfortunately it is not guaranteed to be still optimal for the system with nonwhite or nongaussian gust.

Root Locus for some $q_{22}$ 's and $q_{11}=1.0$. The root loci with varying $r$ are shown in Figs. (6 and 7) for vertical and yaw equations, respectively. The complex roots are circumferently distributed to the origin. These numerical examples are of the following numbers.

Vertical

$$
\begin{align*}
& \mathrm{a}=\mathrm{Z}_{\hat{\mathrm{w}}}=-.2691 / \mathrm{sec} \\
& \mathrm{~b}=\mathrm{Z}_{\delta_{\mathrm{c}}}=-292 \mathrm{ft} / \mathrm{sec}^{2} / \mathrm{rad}  \tag{3-22}\\
& \mathrm{c}=-\mathrm{Z}_{\mathrm{w}}=-.2691 / \mathrm{sec}
\end{align*}
$$

Yawing

$$
\begin{align*}
& \mathrm{a}=\mathrm{N}_{\mathrm{r}}=-.4821 / \mathrm{sec} \\
& \mathrm{~b}=\mathrm{N}_{\delta_{r}}=-8.95 \mathrm{rad} / \mathrm{sec}^{2} / \mathrm{rad}  \tag{3-23}\\
& c=-\mathrm{N}_{\mathrm{V}}=-.0019 \mathrm{rad} / \mathrm{ft} / \mathrm{sec}
\end{align*}
$$

Notice that the closed loop system is sufficiently stabılized by weightıng $q_{11}$ and $r$ only, and the damping ratio approaches .707 as $r$ becomes smaller. These poles are called 2nd order Butterworth poles (Ref. 9 ). Addition of $q_{22}$
gives rise to a more damped systen. The damping ratio and undamped natural frequency are given as:

$$
\begin{align*}
\zeta & =\frac{1}{2} \sqrt{2+\frac{a^{2}}{b} \sqrt{\frac{r}{q_{11}}}+b q_{22} \sqrt{\frac{1}{r q_{11}}}}  \tag{3-24}\\
\omega_{n} & =\sqrt{b \sqrt{\frac{q_{11}}{r}}} \tag{3-25}
\end{align*}
$$

The latter determanes the system response speed, or accuracy, which is inversely proportional to quartic root of control weighting $r$. When the weighting $q_{22}$ satisfies the following inequality

$$
\begin{equation*}
\sqrt{\frac{q_{11}}{q_{22}}} \leqq|a| \tag{3-26}
\end{equation*}
$$

Then all roots are on negative real axis, since $S_{1}=-\sqrt{\frac{q_{11}}{q_{22}}}$ has the role of a zero in the root locus. It is observed that such selection of weightings as in this case 'that $1 s^{\prime}$ no cross product terms in Eq. (3-16), place the closed loop system poles in certain limited left half plane. A desirable region of pole location closely relate with the requirements (2) and (3). In spite of increasing accuracy or speed of response, placing poles too far away in the left half plane, by increasıng feedback gains or control action, is likely to excite the structural vibration mode of the helicopter. On the other hand, poies near the origin present a slow response and roughness of control.

The feedback gains are shown in Figs. (8) and (9), which also show the corresponding augmented damping and stiffness. The feedback gains increase as $r$ becomes smaller, and increase in $q_{22}$ raises the rate feedback gains.

The optimal feedback controls for both vertical and yawing systems do not yield a system which has an overshoot for a step input. Hence, some of our design objectives (stable, no overshoot) are definitely obtained. ISE and ISU Evaluation for Command Input. In general, the ISE and ISU for the second order system to a unit command step input are given as

$$
\begin{align*}
& I S E=\frac{1}{\omega_{\mathrm{n}}}\left(\zeta+\frac{1}{4 \zeta}\right) \\
& =\left.\int_{0}^{\infty} x^{2} d t\right|_{x(0)=1.0} ^{x(0)=0.0}  \tag{3-27b}\\
& \text { ISU }=\frac{\omega_{n}\left(a^{2}+\omega_{\Pi}^{2}\right)}{4 \zeta}  \tag{3-28a}\\
& =\left.\int_{0}^{\infty} \delta^{2} \mathrm{dt}\right|_{\left\lvert\, \begin{array}{l}
x(0)=1.0 \\
\dot{x}(0)=0
\end{array}\right.} \tag{3-28b}
\end{align*}
$$

where
$\omega_{n}:$ closed loop undamped natural frequency rad/sec
$\zeta:$ closed loop damping ratio
$a$ : open loop damping
which are sketched as a function of $\zeta$ in Figure 10 where the minimum ISE is attanned when $\zeta=0.500$, and the ISU is inversely proportional to $\zeta$. In the s-plane the curves for ISE $=$ const, and ISU $=$ const. are shown in Figure (18).

Using Eqs. (3-24), (3-25), (3-27) and (3-28), the integrated square errors and controls to command step input are computed with varying $r$, some $q_{22}$ 's and $q_{11}=1.0$ as shown in Figs. (11) and (12). The ISE value approaches the minimum as $r$ goes zero, while the ISU goes to infinity. The minimum ISE value of the optimal control system is given by substituting Eqs. (3-24) and (3-25) into Eq. (3-27),

$$
\begin{equation*}
\text { Min. ISE }=\frac{1}{2} \sqrt{\frac{q_{22}}{q_{11}}} \tag{3-29}
\end{equation*}
$$

at $x=0$ where the ISU becomes infinlty. This value would become criterion for selection of $q_{11}$ and $q_{22}$, known as that of the model equation given by Eq. (D2-21) in Appendix D. For reference of ISE values, it is desirable to compare with the ISE of the first order system of which command response is easily depicted in mind. Therefore, the following time constant may become an index:

$$
\begin{equation*}
T_{e}=\text { time constant of the first order system with same ISE } \tag{3-30}
\end{equation*}
$$

which is labeled in Figs. (10), (11), (12), (38) and (63). Similarly, one may define the following equation as mean control acceleration.

$$
\begin{equation*}
U_{\mathrm{m}}=\sqrt{\frac{1}{\mathrm{~T}_{\mathrm{e}}} \mathrm{ISU}} \tag{3-31}
\end{equation*}
$$

which would be a rule of thumb for the upper limit of ISU. It is interesting to see that ISE of the optimal control system is a little bit larger than the true minimum ISE as shown in Figure 10.

Gust Response. The requirement for gust response supercedes that for command input in position keeping such as hovering. In addition, knowledge of the gust response is important since the optimal control is obtained independently of the random gust. The assumed gust model is of the following power spectrum density (p.s.d.).

$$
\begin{equation*}
S_{r r}(\omega)=\frac{2 \sigma^{2} d}{d^{2}+\omega^{2}} \tag{3-32}
\end{equation*}
$$

where
$d:$ bandwidth of p.s.d. of the random gust
$\sigma:$ rms value of random gust

The numerical examples are given by

$$
\begin{align*}
& \sigma=10 \mathrm{ft} / \mathrm{sec}  \tag{3-33}\\
& \mathrm{~d}=.314,1.0,2.0 \mathrm{rad} / \mathrm{sec}
\end{align*}
$$

which is a considerable gusty condition.
The rms position error and control are calculated with varying $r$ as shown in Figs. (13) to (16) for both vertical and yawing equations. From these figures, the gust response would be alleviated by increasing $q_{22}$, or increasing rate feedback (see Figs. 8,9), with less control effort. It is much more reduced by decreasing $r$ 'that is' increasing rms control action. As $x$ goes to zero, the rms control force (acceleration) increases to cancel the random gust force with little excess as shown in Figs. (13) and (14). Note that excess control action is caused by control system time lag.

For different bandwidths of p.s.d. of random gust under its constant intensity, the rms position errors and controls are studied in Figs. (15) and (16). These results show that random gust with higher frequency component bring less effect on both rms position error and resulting control.

These results are explained by the frequency domain analysis as follows: The rms value of the gust response is described as (Appendix Gl-3)

$$
\begin{equation*}
\text { RMS value }=\frac{1}{\pi} \int_{0}^{\infty}|G(j \omega)|^{2} S_{r r}(\omega) d \omega \tag{3-34}
\end{equation*}
$$

where $G(j \omega)$ is the frequency response of the transfer function to the gust. For example, the frequency responses of the transfer function of vertical position error and control to the gust input are shown in Figs. (17) and (18). The following three cases are pictured here:

Weighting factors
Case No.
(2)
(3)
$1.0 \quad 0$
$1.0 \quad 4.0 \quad 8190$
1.9

Undamped

| Case No. | $\mathrm{q}_{11}$ | $\mathrm{q}_{22}$ | r | Damping | Natural Frequency |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| (1) | 1.0 | 0 | 13130 | 0.71 | 0.9 |  |
| $(2)$ | 1.0 | 0 | 8190 | 0.71 | 1.8 | $(3-35)$ |
| $(3)$ | 1.0 | 4.0 | 8190 | 1.9 | 1.8 |  |

The broken lines indicate the p.s.d. of the random gust. The ms position error in case (I) becomes larger than the others because of the higher gain as shown in Fig. (17). On the other hand, the control action becomes smaller since the relative magnitude of control is smaller. In case (3), though the control response can be influenced by a high frequency component of random gust, the gust under consideration does not include such frequency component, and therefore the rms resulting control is smaller compared to case (2). Similariy, it is recognyzed that the random gust with higher bandwidth of its power spectrum density has less effect on both rms position errors and control.

Now, assume that requirements are chosen as follows:

$$
\begin{align*}
& \text { rms vertical position error } \leq 5 \mathrm{ft}  \tag{3-36a}\\
& \text { Integral squared position error } \leq 5 \mathrm{ft}^{2} \sec \left(\mathrm{~T}_{\mathrm{e}}=1.0\right) \quad(3-36 \mathrm{a}) \\
& \text { Integral squared control } \leq 9.0 \mathrm{ft}^{2} \sec ^{-3}\left(\mathrm{U}_{\mathrm{m}}=3.0 \mathrm{ft} / \mathrm{sec}^{2}\right) \tag{3-36c}
\end{align*}
$$

Eq. (3-36a) would supercede the others in keeping position.
The lower limit of $r$ is prescribed by only ISE requirement. We should choose $q_{11}$ and $q_{22}$ as necessary,

$$
\begin{equation*}
q_{22} / q_{11} \leq 1.0 \tag{3-37}
\end{equation*}
$$

and $x$ should satisfy the following inequality
$1200(8.5) \leq \frac{r}{q_{11}} \leq 2,200(6.2)$
for $q_{11}=1.0$ and $q_{22}=0.0$.
The requirements for yawing cases are chosen as follows:
rms yawing error $\leq 1.0$ degrees
ISE

$$
\begin{equation*}
\leq .50 \mathrm{deg}^{2} \mathrm{sec}\left(\mathrm{~T}_{\mathrm{e}}=1.0 \mathrm{sec}\right) \tag{3-39a}
\end{equation*}
$$

ISU $\leq 9 . \mathrm{deg}^{2} \mathrm{sec}^{-3}\left(\bigcup_{\mathrm{m}}=30 \mathrm{deg} / \mathrm{sec}\right)(3-39 \mathrm{c})$
Then, we should choose as necessary

$$
\begin{equation*}
q_{22} / q_{11} \leq 1.0 \tag{3-40}
\end{equation*}
$$

and $r$ should satisfy, for $q_{11}=1.0, q_{22}=0$

$$
\begin{equation*}
1.1(8.5) \leq r / q_{11} \leq 3(5.2) \tag{3-41}
\end{equation*}
$$

Note that numerical numbers in the parenthesis indicate ratios of expected controls and position errors due to weightings, and the ratios are almost the same in both cases. There is a big difference between selected $r$ due to the differences in control derivatives.

The desirable root location to meet all requirements (Eq. 3-36), which varies as requirements alternate, is sketched in Fig. (18).

Owem Design Method. The optimal control is given by, as shown in Appendix E2.

$$
\begin{equation*}
\bar{\delta}_{o p t}(t)=-\frac{b \frac{1}{3}}{r \frac{2}{3}} x(t)-\left(-\frac{a}{b}+\sqrt{\left(\frac{a}{b}\right)^{2}+\frac{3}{b^{\frac{2}{3}} r^{\frac{2}{3}}}}\right) \dot{x}(t) \tag{3-42}
\end{equation*}
$$

Note that the optimal control is determined only by $x$ which is called 'trade off parameter' in the Appendix E. This optimal control is also designed to be optimal for the system with a gaussian white noise as shown in the Appendix E, but not for a nonwhite gust.

The close loop system is given by, (See Eq.E2- )

$$
\begin{equation*}
\ddot{:} \dot{x}(t)+\sqrt{a^{2}+\frac{3 b \frac{4}{3}}{r \frac{2}{3}}} \dot{x}(t)+\frac{b \frac{4}{3}}{r \frac{2}{3}} x(t)=0 \tag{3-43}
\end{equation*}
$$

of which undamped natural frequency and damping ratio are:

$$
\begin{align*}
& \omega_{\mathrm{n}}=\frac{\mathrm{b} \frac{2}{3}}{r \frac{1}{3}}  \tag{3-44}\\
& \zeta=\frac{1}{2} \sqrt{3+\frac{a^{2}}{b \frac{2}{3}} \times \frac{2}{3}} \tag{3-45}
\end{align*}
$$

Root Locus for Varying r

The root loci are-sketched in broken lines for vertical and yawing equation in Figs. (6) and (7), respectively. The closed loop system approaches the system with damping ratio $=.866$ while the poles are pushed away
in the left half $s$ plane as $r$ goes to zero. The distance of the complex roots from the origin is inversely proportional to cubic root of control welghting $r$. By this method, no overshoot and sufficiently stable system can be obtaıned and some of our design objectives are achieved.

## ISE and ISU for Command Input

The ISE and ISU values are shown in broken lines in Figs. (11) and (12), respectively. These(ISE and ISU with varying r)are calculated by substituting (3-44) and (3-45) into (3-27) and (3-28) and as $r$ goes zero, obviously

$$
\begin{align*}
& \operatorname{ISE}_{\mathrm{r}=0}=0  \tag{3-46}\\
& \mathrm{ISU}_{\mathrm{r}=0}=\text { infinity } \tag{3-47}
\end{align*}
$$

The requirements described in Eqs. $(3-36 b, c)$ and $(3-39 \mathrm{~b}, \mathrm{c})$ would be satisfied by choosing r properly.

## Gust Response

The rms position exrors and controls of the gust responses are also shown with broken lines in Figs. (13) and (14). Decreasing a weight $r$ on control or increase in available control reduces the rms position error.

The requirements described in Eqs. (3-36) and (3-39) are satisfied by choosing

$$
\begin{align*}
& 2700(5.6) \leq x \leq 7500(3.4)  \tag{3-48}\\
& 2.6(5.6) \leq x \leq 5.8(3.5) \tag{3-49}
\end{align*}
$$

Note that the numbers in parenthesis indicate the ratio of expected controls and state errors due to $r$ called tradeoff parameter in the single input cases [see Appendix E].

In this way, all the design requirements would be satisfied by this method.

## Helicopter in Approach Phase

The numerical values for the helicopter in the approach phase (Eq. 3-14), are not sufficiently different from those in Eq. 3-22 and 3-23 to employ a control system with variable gain, especially as the control system is of the first order.

## C Controller Design for Longitudinal Equation

A hovering helicopter can have an unstable, oscallatory mode in the longitudinal equation, as shown in Figure 2. Furthermore, the position control equation has another pole at origin. That places much workload on the p1lot for precision hovering, especially in gusty conditions. Therefore, the design objectuves become, for the automatic controller:

1. To obtain the stable system
2. Good response, such as little-overshoot, minlmızing the ISE, IS $\theta$, ISU to a command input.
3. To manımize RMS of the position error, attitude error, and resulting control to gust input

The longitudinal equation is given by

$$
\begin{array}{r}
\dot{u}-x_{u}^{u}-X_{q} \dot{\theta}+g \theta=X_{B 1 s} B_{1 s}-X_{u} u_{g} \\
-M_{u} u+\ddot{\theta}-M_{q} \dot{\theta}=M_{B 1 s} B_{1 s}-M_{u}^{u} g \tag{3-50}
\end{array}
$$

In the state variable form, including position variables
where

| x | longitudinal position , ft |
| :---: | :---: |
| u | longitudinal velocity , ft/sec |
| $\theta$ | pitch attitude angle , rad |
| q | : pitch rate , rad/sec |
| $\mathrm{B}_{1 s}$ | longitudinal cyclic pitch , rad |
| $u_{g}$ | : gust input , ft/sec |
| Xu | rate of change of longitudinal force, $1 / \mathrm{sec}$ with velocity |
| Xq | rate of change of longitudinal force, ft/sec with pitch rate |
| Mu | rate of change of pitching moment, rad/ft sec with velocity |
| Mq | : pitch rate damping , 1/sec |
| $\mathrm{X}_{\mathrm{Bls}}$ | longitudinal control derivatives , ft/ $\mathrm{sec}^{2} / \mathrm{rad}$ |
| $M_{\text {Bls }}$ | longitudinal control derıvatives, $\mathrm{rad} / \mathrm{sec}^{2} / \mathrm{rad}$ |
| g | : gravitational force $\quad, 32.2 \mathrm{ft} / \mathrm{sec}^{2}$ |

The performance index to be minımized is given as follows:

$$
\begin{equation*}
\frac{1}{2} \int_{0}^{\infty}\left(q_{11} x^{2}+q_{22} u^{2}+q_{33} \theta^{2}+q_{44} \dot{\theta}^{2}+r B_{1 s}^{2}\right) d t \tag{3-52}
\end{equation*}
$$

Selection of $\mathrm{q}_{\mathrm{ij}}(i=1,2,3,4)$ and $r$.
The selection of diagonal $Q$ or $q_{i i}$ places the poles of the closed loop system in a restricted part of the left half s plane. (Appendix E). The following units are selected for $q_{11}, q_{22}, q_{33}, q_{44}$ and $r$.

$$
\begin{array}{ll}
\mathrm{q}_{11} & \mathrm{ft}^{-2} \\
\mathrm{q}_{22} & \mathrm{ft}^{-2} \mathrm{sec}^{2} \\
\mathrm{q}_{33} & \mathrm{rad}^{-2}  \tag{3-53}\\
\mathrm{q}_{44} & \mathrm{rad}^{-2} \mathrm{sec}^{2} \\
\mathrm{r} & \mathrm{rad}^{-2}
\end{array}
$$

Their values are specified by choosing an equivalence of $x, u, \theta, q$ and $B_{1 s}$ in the integral of Eq. (3-52). The selection is as follows:
$1 \mathrm{ft}=1 \mathrm{ft} / \mathrm{sec}=1$ degree $=1$ degree $/ \mathrm{sec}=0.1 \mathrm{rad}$

Equation (3-54) is expressed as Case No. (7) with $r=100$ in Table 3.2 where the selected diagonal weighting $q_{i i}(i=1,2,3,4)$ are shown. Note that attitude errors $\theta, \dot{\theta}$ are specified in degrees, while in the equation the units are radians.

Table 3-2. Numerical Examples for $q_{i i}$

| Case No. | $q_{11}$ | $q_{22}$ | $q_{33}$ | $q_{44}$ |
| :--- | :--- | :--- | ---: | ---: |
| $(1)$ | 1.0 | $\cdots$ | $\cdots$ | $\cdots$ |
| $(2)$ | 1.0 | $\cdots$ | 820 | $\cdots$ |
| $(3)$ | 1.0 | $\cdots$ | 3,280 | $\cdots$ |
| $(4)$ | 0.1 | $\cdots$ | 3,280 | $\cdots$ |
| $(5)$ | 0.01 | $\cdots$ | 3,280 | $\cdots$ |
| $(6)$ | 1.0 | $\cdots$ | 3,280 | 3,280 |
| $(7)$ | 1.0 | 1.0 | 3,280 | 3,280 |
| $(8)$ | 0.1 | 0.1 | 3,280 | $\cdots$ |
| $(9)$ | 0.1 | 0.1 | 3,280 | 3,280 |
| $(10)$ | 0.1 | 0.4 | 3,280 | 13,130 |

## Root Square Locus.

The root loci for the longitudinal system is shown in Figs. (19) through (27). A stable system is obtained using only a weighting $q_{11}$ of the $Q$ matrix and $r$. Two of the poles approach the asymptote $\zeta=0.707$ while attitude poles approach the zeroes, as shown in Fig. (19). The root loci starts from stable poles which are then shifted to mirror image in the left half plane of the orlginal unstable poles. Increasing the attitude weighting factors $q_{33}$ from a zero value, induces the more damping to the attitude poles, and further increases in $q_{33}$ moves the attitude poles away from the asymptote and shifts the position poles into the left half plane as shown in Figs. (20) to (23). As a result, the att1tude control or inner loop control becomes much faster than the outer loop or position control. The boundary condition, whether or not the locus of the attitude poles bends inwards, depends on the weighting ratio of $q_{11}$ and $q_{33}$. The zero locus is drawn as a function of $q_{33} / q_{11}$ in Fig. (22). This indicates that an increase in the relative weighting $q_{33}$ to $q_{11}$ brings two zeroes to the origin. Addition of either $q_{22}$ or $q_{44}$ yields another zero on real axis and provides more damping to both the attitude and position poles, as shown in Figs. (24) to (27). Especially, imposition of $q_{33}$ on $\dot{\theta}$ locate a zero near $z_{1}=-\sqrt{\frac{q_{33}}{q_{44}}}$. Hence, only one root goes to infinity along the negative real axis, while the others approach the zeroes, and the system is more damped. The closed loop system with damping ratio more than 0.707 would be achieved by the adequate selection of $q_{11}, q_{22}, q_{33}, q_{44}$ and $r$, where obviously more weighting would be imposed on $q_{33}$ and $q_{44}$.

By the OWEM method, the root locus is uniquely determined with choice of $r$ only. Four poles move to infingty along $\zeta \approx .65$ and $\zeta \approx .95$, and the closed loop systems are sufficiently damped.

In this way, some of the design requirements are attained without difficulty. The feedback gains are shown in Figs. (28) to (37). Using these diagrams, the optimal control is obtained as:

$$
\begin{equation*}
\mathrm{B}_{1 \mathrm{~s}}=K_{x} x+K_{u} u+K_{\theta} \theta+K_{q} q \tag{3-55}
\end{equation*}
$$

At infinite $r$, both feedback gains $K_{\theta}$ and $K_{q}$ approaches nonzero gains, so that the unstable poles are shifted to the mirror amage in the left half plane. For the conventional selection method of $q_{i i}$ and $r$, the outer loop, feedback gain is gıven by:

$$
\begin{equation*}
K_{x}=\sqrt{\frac{q_{11}}{r}} \tag{3-56}
\end{equation*}
$$

which specifies the systems accuracy. From this point of view, Eq. (3-56) helps select $q_{11}$ and $r$.

ISE, IS $\theta$ and ISU for a Command Input.
The integral squared position and attitude error to a 10 ft . command input is shown in Fig. (38). In position control, the hovering helicopter is desirable to follow the command input as fast as possible without excessive attitude change or 1 ts rate. As r goes to zero, the ISE value approaches the minimum given by that of model equation corresponding the zeroes, while the IS $\theta$ reaches the maximum as shown in Fig. (38). These values at null $r$ are sketched as a function of ratio of weighting factors $q_{11}$ and $q_{33}$ in Fig. (39).

They show that the reduction in ISE increases IS $\theta$ in Fig. (39). Hence, a compromase is necessary. The addition of the weighting factor, $q_{44}$ decreases the IS $\theta$ considerably, with little increase in the ISE. Finally, the amount of control available in Fig. (40) is limited by the power limitation and mechanical strength of the helicopter.

As shown in Fig. (38), the integral squared control increases as $r$ decreases and decreases more when a weighting is placed on the attitude or its rate.

It is observed that the OWEM has features similar to, but better than, Case 1 in Table 2, indicating ISE $=0$, IS $\theta=$ ISU $=\infty$ at $\mathrm{r}=0$.

## Gust Response

The rms position and attitude errors to the random gust are shown in Figs. (41) and (42), for some $q_{i i}$ 's and varying r. The assumed gust model is of the same as Eq. (3-33) and numerical example is made assuming quite strong gusty conditions.

$$
\begin{aligned}
& \sigma=20 \mathrm{ft} / \mathrm{sec} \\
& \mathrm{~d}=.314,1.0,2.0 \mathrm{rad} / \mathrm{sec}
\end{aligned}
$$

The rms position error decreases as $r / q_{11}$ becomes smaller and depends slightly on $q_{33} / q_{11}$. Effects of $q_{22}$ and $q_{44}$ are negligible, and hence, not shown in the figure. On the other hand, the rms attitude error, due to the gust, depends strongly on $r / q_{11}$ and selected $q_{i i}(i=1,2,3,4)$. Addıng a weighting factor, $q_{22}$ or $q_{44}$ improves the error considerably. In the range of $q_{i i}$ considered here, the ratio of $q_{33} / q_{11}$ governs entirely the order of magnitude of the rms value.

The rms longitudinal cyclic control and position error is shown in Fig.(41). The root mean squared position, attitude errors, and different bandwidths of the power spectrum density for a gust with constant intensity are computed. It is clear that the rms position errors are less influenced by the gust with higher bandwidth. On the other hand, the rms attitude errors are affected less at a large $r$, but become more affected at smaller $r$ by the gust components of higher frequency, as shown in Fig. (67).

The gust responses for the OWEM, shown with broken lines in Figs. (41) and (42) are better than Case 1.

A ranking of all requirements for precision hover can be done as shown below. A compromise between them nust be found as it is desirable that all requirements are simultaneously satisfied. Take, for example, the following case:

| root mean squared position error | $\leqq$ | 1.0 ft. |
| :--- | :--- | :---: |
| root mean squared attitude error | $\leqq$ | 0.3 deg. |
| integral squared position error | $\leqq 400 \mathrm{ft}^{2} \mathrm{sec}(\tau \mathrm{s} \leq 8)$ |  |
| integral squared attitude | $\leqq$ |  |
| integral squared control | $\leqq 0 \mathrm{deg}^{2} \mathrm{sec}(\theta \mathrm{m} \leq 2 \mathrm{deg})$. |  |
|  | $\leqq 0.05 \mathrm{rad}^{2} \mathrm{sec}\left(\mathrm{B}_{1 \mathrm{sm}} \leq .05 \mathrm{rad}\right)$ |  |

acceleration limits $\leqq$ none
The righthandside of Eqs. (3-57a) and (3-57b) vary depending on rms gust velocity or its rate. The upper limits on Eqs. (3-57c) to (3-57e) are altered proportionally to squared magnitudes of command inputs. However,
$T_{e_{\max }}, \theta_{\max }$, and $U_{\max }$ may be independently prescribed.
Equation (3-57a) exclusively determines the upper limits of $r$, though
ISE and rms attitude error requirements may influence this considerably. The lower limit of $x$ is described by Eqs. (3-57d) and (3-57e). The way of choosing $q_{i i}$ could be found out by requirements (3-57b), (3-57d), and (3-57e). If one considers the above requirements, then the following set of $q_{i i}$ and $r$ can be chosen as one of desirable sets of $q_{i i}$ and $r$.

$$
\begin{align*}
& r=2.0 \cdot 10^{4} \\
& q_{11}=.1, q_{33}=3280 \tag{3-58}
\end{align*}
$$

With the feedback gains

$$
\begin{align*}
& K_{x}=-.0071 \\
& K_{u}=-.026  \tag{3-59}\\
& K_{\theta}=1.58 \\
& K_{q}=.56
\end{align*}
$$

It is easily understood in the process of choosing these sets that Case (5) does not satisfy Eq. (3-57c) and Case (1) is unsatisfactory_with Eq. (3-57d), because of excessive attitude error. An OWEM method which does not take into account the difference in position and attitude error weighting should also be unsatisfactory (for example, see Eq. (3-57d).

## Helicopter in Approach Phase

The numerical values for the helicopter in the approach phase are given in Eq. (3-14). The root location for the optimal roots as a function of the control weighting $r$ are shown in Figure 67. It appears that the difference in numerical values compared to the hovering case is not sufficiently large enough to justify a control system with variable gains.

## OWEM Design Method

Applying the OWEM design method for the longitudinal motion of the helicopter gives a root location as a function of $r=\rho$ obtained by a computer solution as shown in Figure 19.

In the initial approach of an OWEM design, all the constants $k_{3}$ of the group-wise equalization of the terms of $\left\{y *{ }^{T} Q y *\right\}$ are chosen to be equal. This corresponds to an equalization matrix $S=I$. Making the equalization matrix not equal to the identity matrix corresponds to an equalization factor of the diagonal terms of the matrıx $q_{i j}$. This principle is illustrated in this example in Figure 19.

In this figure is shown the root location obtained by the initial application of the OWEM method (equalization matrix = identity matrix). The desired root location with associated optimal $Q$ matrix is determined by the acceleration limitation of the helicopter (corresponding to an $\omega$ of .3 to $5 \mathrm{rad} / \mathrm{sec}$ ). Take, for example, that an equalization was desired of one to two for the ratio of the position error to attitude error. The diagonal terms of the $Q$ matrix are associated with the variables $x, \dot{x}, \theta, \dot{\theta}$ respectively. Hence, the equalization is chosen to be $1: 1: 4: 1$.

Figure 69 is shown, the root location obtained by the OWEM method, modified with an error scaling matrix (ESM) of 1:1:4:1. For this ESM Q matrix a root square locus diagram is given as a function of the control weighting $\rho$. The ESM matrix should also change the weighting of the derivatives of $\theta$, as this increases the damping. By trial and error, one can investigate the error scaling matrix for the $\dot{\theta}$ error weighting. In this particular case, 1:1:4:4 was chosen.

In the same illustration is shown the chosen optimal $Q$ matrix, modified with an equalization matrix of 1:1:4:4.

For this equalized $Q$ matrix a root square locus diagram is given as a function of the control weighting $r=\rho$. (curve no. 2)

For a control weighting of 3200 , the (negative) feedback gains are:

$$
\begin{align*}
& \mathrm{G}_{\mathrm{x}}=-.0015 \mathrm{rad} / \mathrm{ft}=-.084 \mathrm{deg} / \mathrm{ft} . \\
& \mathrm{G}_{\dot{\mathrm{x}}}=-.0074 \mathrm{rad} / \mathrm{ft}=-.420 \mathrm{deg} / \mathrm{ft} . \\
& \mathrm{G}_{\theta}=.56 \mathrm{rad} / \mathrm{rad}=.56 \mathrm{deg} / \mathrm{deg} .  \tag{3-60}\\
& \mathrm{G}_{\dot{\theta}}=.42 \mathrm{rad} / \mathrm{rad}=.42 \mathrm{deg} / \mathrm{deg} .
\end{align*}
$$

The root locations are shown in Figure 69, with the associated feedback gains in Figure 72. The corresponding error and control measures are shown in Figures 40, 41 and 42.

It should be emphasized that, in this work the acceleration limitations have been introduced as an "afterthought" in the OWEM design. In the OWEM calculations the acceleration limıtations were simulated by an "error weighting". In principle, it should be possible to introduce constraints directly into the OWEM design method, thereby eliminating the trial and error aspect.

In the hovering helicopter example, feedback gain limntations on the position state variable (equivalent to an acceleration limit) should more or less "freeze" the position mode when a certain control power is reached. A further increase in control power should only affect the $\theta$ mode. This procedure would eliminate the trial and error procedure of the $\dot{\theta}$ error welghting. How~ ever, these theoretical aspects need first to be further investigated.

In a hovering helicopter, the lateral equation uncoupled with yaw becomes unstable and poses a comparatively short period mode (see Figs. (2,3)). Therefore, our design objectives are composed of the same ones as the longitudinal case. The lateral equations are given by:

$$
\begin{array}{r}
\dot{v}-Y_{v} v-Y_{p} p-g \phi=Y_{A 1 s} A_{1 s}-Y_{v} v_{g} \\
-L_{v} v+\ddot{\phi}-L_{p} \dot{\varphi}=L_{A 1 s} A_{1 s}-L_{v} v_{g} \tag{3-61}
\end{array}
$$

In the state variable form, including position variable

$$
\left[\begin{array}{c}
\dot{y} \\
\dot{v} \\
\dot{\phi} \\
\dot{p}
\end{array}\right]=\left[\begin{array}{cccc}
0 & 1 & 0 & 0 \\
0 & Y_{V} & g & Y_{p} \\
0 & 0 & 0 & 1 \\
0 & L_{v} & 0 & L_{p}
\end{array}\right]\left[\begin{array}{l}
Y \\
v \\
\phi \\
P
\end{array}\right]+\left[\begin{array}{l}
0 \\
Y_{\text {Als }} \\
0 \\
L_{A l s}
\end{array}\right] A_{1 s}+\left[\begin{array}{c}
0 \\
-Y_{V} \\
0 \\
-L_{v}
\end{array}\right] v_{g} \quad(3-62)
$$

where

$$
\begin{array}{lll}
y & : & \text { lateral position } \\
v & : & \text { lateral velocity } \\
\phi & : \text { roll angle } & , f t / s e c \\
p & : \text { roll rate } & , \text { rad } \\
& \text { radsec }^{-1}
\end{array}
$$

| $\mathrm{A}_{15}$ | : lateral cyclic patch | , rad |
| :---: | :---: | :---: |
| $\mathrm{v}_{\mathrm{g}}$ | : gust input | ,ft/sec |
| $Y_{v}$ | : rate of change of lateral force with velocity | , $\sec ^{-1}$ |
| $Y_{p}$ | : rate of change of lateral moment w1th roll velocity | , $\sec ^{-1}$ |
| $\mathrm{L}_{\mathrm{V}}$ | : rate of change of lateral moment with velocity | , rad/sec ft |
| $L_{p}$ | : roll rate dampıng | , $\sec ^{-1}$ |
| $\mathrm{Y}_{\text {Als }}$ | : lateral control derivatives, | ,ft. $\mathrm{sec}^{-2} \mathrm{rad}$ |
| $\mathrm{L}_{\text {A1s }}$ | : lateral control derivatives, | , $1 / \mathrm{sec}^{2}$ |
| g | : gravitational force | ,32.2 $\mathrm{ftsec}^{-2}$ |

The performance index to be minımızed is given by:

$$
\begin{equation*}
\int_{0}^{\infty}\left(q_{11} y^{2}+q_{22} v^{2}+q_{33} \phi^{2}+q_{44} p^{2}+r A_{1 s}^{2}\right) d t \tag{3-63}
\end{equation*}
$$

Selection of $q_{i 1}, i=1,2,3,4$ and $x$.
The diagonal weighting matrices $Q$ might place the poles of the closed loop systems in restricted parts of the left half plane. The selection of Q would be done in the same way as in the longitudinal case and the same $\mathrm{q}_{\mathrm{ii}}$ 's are used as numerical examples [see Table 3.2].

The root loci are shown in Figs. (43) to (51). The stable closed loop system is obtained by a weighting factor $q_{11}$ only. Addition of $q_{33}$ gives more damping to the system, especially to attitude control loop. Further increase in $q_{33}$ finally moves the poles of the attitude loop further in the left half plane, while the poles of position move to the right, 'that is' the position loop reduces stability as shown in Figs. (45 to (47). It is seen in Fig. (46) that the attitude poles start going far away for a set of $q_{11}=.1, q_{33}=3280$ unlike the longitudinal case. These tendancies seem desirable since the inner loop control is desirable to be much faster than the outer loop position control. The broken line of Fig. (44) indicates the zero locus as a function of the parameter of $q_{33} / q_{11}$. It starts from the point $\zeta=-2.33 \pm j 4.06$ and approaches the zero along the asymptote $\zeta=.707$. Addition of $q_{22}$ or $q_{44}$ yields another zero on negative real axis and provides more damping to both the attitude and position poles as shown in Figs. (48, 49, 50 and 51). Selection of $q_{22}$ locates the zero in the far left of the negative real axis. On the other hand, selection of $q_{44}$ locates the zero on $z_{1}=-\sqrt{\frac{q_{33}}{q_{44}}}$. As a result, only one root goes to infinity on the negative real axis while the others approach zeroes and the system is more damped. The closed loop system with damping ratio more than 0.707 , if desired, would be achieved by adequate selection of $q_{11}, q_{22}, q_{33}, q_{44}$ and $r$.

The feedback gains are shown in Figs. (52) to (61), where it is noted that the augmented dampings are equivalent to those of longitudinal ones at the same $r / q_{11}$, though feedback gains being smaller than longitudinal ones
except $\mathrm{K}_{\mathrm{y}}$ given by

$$
\begin{equation*}
K_{y}=\sqrt{\frac{q_{I 1}}{r}} \tag{3-64}
\end{equation*}
$$

which specify the system control precision. At infinite $r, K_{\phi}$ and $K_{p}$ are nonzero simularly as in longitudinal cases. Using these gain diagrams, the optimal control is obtained as:

$$
\begin{equation*}
A_{1 s}^{*}=K_{y} y+K_{v} v+K_{\phi} \phi+K_{p} p \tag{3-65}
\end{equation*}
$$

ISE, IS $\phi$, and ISU for Command Input

The integral squared position error, attitude and control to a 10 ft . command input is shown in Figs. (62) to (63). Effects of $r$ and $q_{i i}$ on these results are almost the same as the longitudinal case. When compared to the longitudinal case, the IS $\theta$ came out larger in spite of the use of less control amount (ISU) due to the larger control derivative. Accordingly, the command input response becomes faster, that 15 , less ISE. The min. ISE and $\max$. IS $\theta$ at zero $r$ are nearly equivalent to those of the longitudinal case because both model equations are almost the same. Note that the ISE goes zero while the IS $\theta$ increases to infinity as r goes zero for the OWEM. Gust Responses

Figs. (64) to (67) show that rms lateral position error (ISE), attıtude error (ISゆ) and resulting control (ISU). From these figures, it is confirmed that the lateral motion is much stronger influenced by gust through large gust derivatives, $Y_{V}$ and $L_{v}$, compared to longitudinal motion. The control
necessary to trim the translational force and upset moment due to gust amounts is twice as large as that of a longitudinal case. Keeping lateral position as precise as longıtudinal control requires more feedback gains corresponding to smaller r .

Figure (66) indicates the effect of gust with different frequency components under its constant intensity to attitude errors. It turns out that the gust with cut-off frequency $d=1.0 \mathrm{rad} / \mathrm{s}$ influences the lateral attitude most, while the pitch attitude is most influenced with cut-off frequency $d=.314 \mathrm{rad} / \mathrm{s}$ at large r . These characteristics are explanned by the fact that the basic helicopter dynamics have modes excited around frequency of .36 rad . in longitudinal motion and frequency of .85 rad . in lateral axis.

Take, for example, the following case similarly to the Eqs. (3-57a) to (3-57e):

| rms position error | $\leq 1.0 \mathrm{ft}$. | $(3-66 \mathrm{a})$ |
| :--- | :--- | :--- |
| rms roll attitude error | $\leq 0.3 \mathrm{deg}$. | $(3-66 \mathrm{~b})$ |
| ISE | $\leq 400 \mathrm{ft}^{2} \mathrm{sec}\left(\mathrm{T}_{\mathrm{em}} \leq 8 \mathrm{sec}\right)$ |  |
| IS $\theta$ | $\leq 16 \mathrm{deg}^{2} \mathrm{sec}\left(\theta_{\mathrm{m}} \leq 2 \mathrm{deg}\right)$ |  |
| ISU | $\leq 06 \mathrm{c})$ |  |
| Acceleration 1 imits | $\leq 0.05 \mathrm{rad}^{2} \mathrm{sec}\left(\mathrm{A}_{1 \mathrm{sm}} \leq .05 \mathrm{rad}\right)$ |  |
| $(3-66 \mathrm{e})$ |  |  |

The design approach is similar to that of the longitudinal case.
For the closed loop system, to satisfy all requirements, the following set of $q_{i i}$ and $r$ s chosen as one of the desirable sets of $q_{i 1}$ and $r$

$$
\begin{align*}
& r=1.5 \cdot 10^{3}  \tag{3-67}\\
& q_{11}=.1, q_{33}=3280
\end{align*}
$$

Then, the feedback gains are:

$$
\begin{align*}
\mathrm{K}_{\mathrm{y}} & =-.026 \mathrm{rad} / \mathrm{ft} \\
\mathrm{~K}_{\mathrm{v}} & =-.087 \mathrm{rad} / \mathrm{ft} / \mathrm{s}  \tag{3-68}\\
\mathrm{~K}_{\phi} & =-4.8 \mathrm{rad} / \mathrm{rad} \\
\mathrm{~K}_{\dot{\phi}} & =-.46 \mathrm{rad} / \mathrm{rad} / \mathrm{s}
\end{align*}
$$

In a practical application, the acceleration limıts will show that these gains are too high.

## Helicopter in Approach Phase

The numerical values for the helicopter in the approach phase are given in Eq. (3-14). The root location for the optimal roots as a function of the control weighting $\rho$ is shown in Figure 68. It appears that the difference in numerical values, compared to the hovering case, is not sufficiently large enough to justify a control system with varıable gains.

## OWEM Design Method

Using the approach slmilar to that of the longitudinal case, the root locations using the OWEM design is shown in Figure 43. For an error scaling matrix, chosen was $1: 1: 4: 3$, and, the feedback gains for $\rho=1300$ are:

$$
\begin{align*}
& K_{y}=-.00075 \\
& K_{y}=-.0025  \tag{3-69}\\
& K_{\phi}=-.18 \\
& K_{\dot{\phi}}=-.09
\end{align*}
$$

The root locations as a function of $r=\rho$ are shown in Figure 70 with the associated feedback gains in Figure 72. The corresponding errors and con-
trol measures are shown in Figures 62, 63 and 64.

CHAPTER 4
CONCLUSIONS AND RECOMMENDATIONS
The optımal linear-quadratic control theory provides a well organized design procedure for the automatic controller dessgn of a multivariable system. In this design process an optimal linear feedback control law is found, which minımizes a quadratic cost function.

It is usually assumed that the weighting matrıces in quadratic cost functions are given a priori. However, the resulting optimal feedback control system is a function of the selected weighting matrices. The functional relationships between the weighting matrices and dynamical characteristics of the resulting system are rather obscure. Therefore, the selection of weighting matrices is often a major problem in the practical applications of the optimal linear-quadratic control theory.

In this report the weighting matrices are selected by the OWEM design method as described in Ref. [7] and Appendix E. Also in this work the werghting matrices using the OWEM design method are obtained by direct calculation and not mainly by a creative trial and error approach. The OWEM design method is compared with other methods for finding the weighting matrices. The results indicate that the use of this approach gives at least as good results as obtalned by other classical design methods.

Much work still has to be done, especially in the case of multivariable input-output systems, and a priori imposed limitations of the practical system. However, from the examples in this work, it can be recommended that the OWEM
design method is a good candadate to use for the selection of weightang matrices. Moreover, the OWEM design method does not require any lengthy and 1 aborous estimations of weighting matrices in quadratic cost function. A completely computerized system design becomes possible, which can provide quick results even for complicated systems. The decision which has to be made by the designer after the computerized system design is to choose an appropriate trade-off parameter by considering the corresponding feedback gains, system dynamical characteristics and other practical situations. Practically, of course, further refinements for such a computerized system design would have to be done for a few choices of trade-off parameters. In the long run, when the given system is highly complicated, for example, a system with many feedback loops and/or multiple controls, the computerized system design procedure associated with the OWEM method can save large amounts of time usually required in the early stage of system designs.

In conclusion, the OWEM method provides an objective measure to determine the weighting factors in the performance index based upon an error criteria. Initially, the system errors are assumed to be equally important, however, directly from missions requirements one can determine which are the important errors, and use this additional information together with the other system requirements to obtain an mproved design. The resulting OWEM design is a control system that minimizes the errors due to disturbances and has good transient responses.

This work explores the application of the OWEM design method to an automatic controller design for a helicopter at hover and approach. The examples demonstrate that this approach is feasible and is a good candidate for the control system design.

Areas of further studies are:

1. Further investigation of the practıcal aspects of system disturbances in the selection of the weighting matrices, i.e., how are the weighting matrices and the resulting optimal feedback gains changed if the noise characterıstics are varied.
2. Investrgation of the effect of a priori imposed limitations (control and/or system boundaries) on the OWEM design method, i.e., in this work imposed limitations were combined with an assigned error scaling matrix ESM derived from physical limitations. Further investigation of the practical aspects of a priori imposing design limitations would be highly desirable.
3. An investigation of the practical aspects of the effects of an incomplete measurement system and feedback of the state variables in the OWEM design method.
4. Further investigation of the applıcation of the OWEM design method in model following control systems. In theory, the OWEM method can yield the weighting or $Q$ matrix for the prefilter model reference control system. However, the practical aspects should be investigated.
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Figure 1. Block Diagram for Decoupling

System dynamics: $\dot{x}(t)=A x(t)+B u(t)+C n(t)$

$$
y(t)=H x(t)
$$

where $x(t)$ : $n$ dimensional state vector
$u(t): m$ dimensional control vector ( $m \leq n$ )
$y(t): p$ dimensional output vector ( $p \leq n$ )
$n(t): \ell$ dimensional noise vector ( $\ell \leq n)$



















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Figure 3. General Construction
for Decoupling Helicopter Dynamics


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 $H$ －$-1 \rightarrow \square$


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Fig. 17 Frequency response of $\omega_{g}-z$ trensfer function


Fig.l7bFrequency response of $\omega_{g} \rightarrow \theta_{0}$ transfer function (straight inne approximation)




































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 $\rightarrow+\rightarrow \rightarrow+\rightarrow+\rightarrow$ $q_{11}=0.1$
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Fig. 28. Longıtudinal optımal feedback gains for the OWEM method



Fig. 30 Longitudinal ontimal Leedback gains for $q_{11}=1.0, q_{33}=820$


Fig. 31 Longıtudinal ontimal feedback gains for $q_{11}=1.0, q_{33}=3283$ keurthe a esser go


Fig. 32 Longitudinal optimal feedback gains for $q_{11}=0.1, q_{33}=3283$


FIg. 33: Longitudunal optimal feedback gains for $q_{11}=0.01, q_{33}=3283$


Fig. 34 Lontıgudinal optımal feedback gains for $q_{11}=1.0, q_{33}=q_{L L L}=3283$


Fig. 35 Longatudunal optimal feedback gains for $q_{11}=q_{22}=1.0, q_{33}=q_{44}=3283$


Fig. 36 Longitudinal oplimal feedback gains for $q_{11}=q_{22}=0.1, q_{33}=q_{4}=3283$


Fig. 37 Longitudinal optimal feedback gains for $q_{11}=0.1, q_{22}=0.4, q_{33}=3283, q_{44}=\$ 3132$





equation, with varyint, $r$ " $q_{i . i}(i=1,2,3,4)$




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Fig. 48



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 $=q_{22}=1.0, q_{33}=q_{4}=3283$ for lateral equation $=t \rightarrow-t+t$


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Fig. 52. Lateral optimal feedback gains for the OWLM method

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Fig. 54 Lateral oplimal feedback gains for $q_{11}=1.0, q_{33}=820$


Fig. 55 Lateral optimal feedback ganns for $q_{11}=1.0, q_{33}=3283$

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Fig. 56 Lateral optimal feedback ganns for $q_{11}=0.1, q_{33}=3283$


Fig. 57 Lateral optimal feedback gains for $q_{11}=0.01, q_{33}=3283$


Fig. 58 Lateral optimal feedback gains for $q_{11}=q_{22}=0.1, q_{33}=3283$




Fig. 61 Lateral ontimal feedback gains for $q_{11}=0.1, q_{22}=0.4, q_{33}=3283, q_{1 / 4}=13132$









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Figure 71. Longıtudinal Gains OWEM wath ESM (1:1:4:4)


Figure 72. Lateral Gain OWEM with ESM (1:1:4:3)

## APPENDIX AI

REVIEW OF HELICOPTER DYNAMICS

## Introduction

In a synthesis or analysis of a control system of helicopters, we are confronted with the complexity of the overall helicopter dynamics. Therefore, slmplifications often are made and are considered the simplified equathons. However, this can be an oversimplification and when the system is constructed, the resultang system may have undesirable properties.

Although the higher order equation can be manipulated by the computer, the analysis and synthesis cannot be easily done. Also the physical insight into the dynamıcs is often lost during a process of computation. Finally, we would be obliged to use brute forces of computing all parameter variations in order to know therr effects. The synthesized controller may become complex, and often becomes more expensive and less reliable. In practice, the precision of hovering can be diminished because of coupling ef.fects.

In general, the complicated dynamics are due to many coupling terms which come from asymmetrical body and power sources. With the exception of a few terms such as $\left(I_{r}\right.$ and $\left.N_{V}\right)$ which contribute to dynamic stability, they usually bring undesirable effects into the dynamics. These effects can be shown from a sensitivıty analysis on coupling terms.

In order to circumvent these things, the dynamics are required to be decoupled. The control of the helicopter then becomes simpler and easier. The analysis and synthesis can be applied to such conventional techniques such as root locus and frequency methods which basically deal with the single
input-single output system. The dynamics can be evaluated by hand calculation. Even in such modem control techniques as optimal control using state space concepts, the analysis and synthesis technique becomes more tractable. This method will be dealt with in the other appendices.

## Helicopter Dynamics

In general, the helicopter dynamics are written with respect to the stability axis [Ref. Al]. The following small angle condltions are assumed and higher order terms are neglected.

$$
\begin{align*}
& \sin \theta=\theta, \cos \theta=1, \sin \phi=\phi, \cos \phi=1  \tag{A1-1}\\
& \dot{\theta}=q, \dot{\phi}=p, \dot{\psi}=r \\
& \dot{u}+g \cos \gamma_{0} \theta=X_{u} u+X_{v} v+X_{w} w+X_{p} p+X_{q} q+X_{r} r+\Delta X_{c} \\
& \dot{v}-g \cos \gamma_{0} \phi=Y_{u} u+Y_{v} v+Y_{w} w+Y_{p} p+Y_{q} q+\left(Y_{r}-U_{0}\right) r+\Delta Y_{c} \\
& \dot{w}+g \sin \gamma_{0} \theta=Z_{u} u+Z_{v} v+Z_{w} w+Z_{p} p+\left(Z_{q}+U_{o}\right) q+Z_{r} r+\Delta Z_{c} \\
& \dot{p}-\dot{r} \frac{I_{x z}}{I_{x}}=L_{u} u+L_{v} v+L_{w} w+L_{p} p+L_{q} q+L_{r} r+\Delta L_{c} \\
& \dot{q}=M_{u} u+M_{v} v+M_{w} w+M_{p} p+M_{q} q+M_{r}+\Delta M_{c} \\
& \dot{r}-\dot{p} \frac{I_{x z}}{I_{z}}=N_{u} u+N_{v} v+N_{w} w+N_{p} p+N_{q} q+N_{r}+\Delta N_{c}
\end{align*}
$$

where

$$
\begin{align*}
& \Delta X_{c}=X_{B_{1 s}} B_{1 s}+X_{A_{1 s}} A_{1 s}+X_{\delta_{c}} \delta_{c}+X_{\delta_{r}} \delta_{r} \\
& \Delta Y_{c}=Y_{B_{1 s}} B_{1 s}+Y_{A_{1 s}} A_{1 s}+Y_{\delta_{c}} \delta_{c}+Y_{\delta_{r}} \delta_{r} \\
& \Delta Z_{c}=Z_{B_{1 s}} B_{1 s}+Z_{A_{1 s}} A_{1 s}+Z_{\delta_{c}} \delta_{c}+Z_{\delta_{r}} \delta_{r} \\
& \Delta L_{c}=L_{B_{1 s}} B_{1 s}+L_{A_{1 s}} A_{1 s}+L_{\delta_{c}} \delta_{c}+L_{\delta_{r}} \delta_{r}  \tag{Al-3}\\
& \Delta M_{c}=M_{B_{1 s}} B_{1 s}+M_{A} A_{1 s}+M_{\delta_{c}} \delta_{c}+M_{\delta_{r}} \delta_{r} \\
& \Delta N_{c}=N_{B_{1 s}} B_{1 s}+N_{A} A_{1 s}+N_{c} \delta_{c}+N_{r} \delta_{r}
\end{align*}
$$

The new derivatives for cancelling the inertia coupling terms are introduced into the last two equations of (A1-2) [Ref. A2].

$$
\begin{align*}
& -\dot{p}+L_{u}^{\prime} u+L_{v}^{\prime} v+L_{w}^{\prime} w+L_{p}^{\prime} p+L_{q}^{\prime} q+L_{r}^{\prime} r+\Delta L_{c}^{\prime}=0  \tag{Al-4}\\
& -\dot{r}+N_{u}^{\prime} u+N_{v}^{\prime} v+N_{w}^{\prime} w+N_{p}^{\prime} p+N_{q}^{\prime} q+N_{r}^{\prime} r+\Delta N_{c}^{\prime}=0
\end{align*}
$$

where

$$
\begin{equation*}
L_{u}^{-}=\frac{L_{u}+\frac{I_{x z}}{I_{x}} N_{u}}{1-\frac{\left(L_{x z}\right)^{2}}{I_{x} I_{z}}}, \quad L_{v}^{\prime}=\frac{L_{v}+\frac{I_{x z}}{I_{x}} N_{v}}{1-\frac{\left(I_{x z}\right)^{2}}{I_{x} I_{z}}}, \ldots . \text { etc. } \tag{Al-5}
\end{equation*}
$$

$$
N_{u}^{-}=\frac{N_{u}+\frac{I_{x z}}{I_{z}} L_{u}}{1-\frac{\left(I_{x z} I^{2}\right.}{I_{x} I_{z}}}, \quad N_{v}^{\prime}=\frac{N_{v}+\frac{I_{x z}}{I_{z}} L_{v}}{1-\frac{\left(I_{x z}\right)^{2}}{I_{x} I_{z}}}, \ldots \text { etc. }
$$

In the same way,

$$
\begin{align*}
& \Delta L_{c}^{\prime}=L_{B_{1 s}^{\prime}} B_{1 s}+L_{A_{1 s}^{\prime}} A_{1 s}+L_{\delta}^{\prime} \delta_{c}+L_{\delta_{r}^{\prime}} \delta_{r}  \tag{A1-6}\\
& \Delta N_{c}^{-}=N_{B_{1 s}^{\prime}} B_{1 s}+N_{A}^{\prime} B_{l s}+N_{\delta_{c}} \delta_{c}+N_{\delta_{r}^{\prime}} \delta_{r}
\end{align*}
$$

where

$$
\begin{aligned}
& L_{B_{1 s}^{\prime}}^{\prime}=\frac{L_{B_{1 s}}+\frac{I_{x z}}{I_{x}} N_{B_{1 s}}}{1-\frac{\left(I_{x z}\right)^{2}}{I_{x} I_{z}}}, L_{\delta_{c}}^{\prime}=\frac{L_{\theta_{0}}+\frac{I_{x z}}{I_{x}} N_{c}}{1-\frac{\left(I_{x z}\right)^{2}}{I_{x} I_{z}}}, \ldots \text { etc. } \\
& N_{B_{1 s}}^{\prime}=\frac{N_{B_{1 s}}+\frac{I_{x z}}{I_{z}} L_{B_{1}}}{1-\frac{\left(I_{x z}\right)^{2}}{I_{x} I_{z}}}, N_{\delta_{c}}^{\prime}=\frac{N_{\delta_{c}}+\frac{I_{x z}}{I_{z}} L_{\delta}}{1-\frac{\left(I_{x z}\right)^{2}}{I_{x} I_{z}}}, \ldots . \text { etc. }
\end{aligned}
$$

In hovering, the helicopter dynamics are desirable to be decoupled into the longitudinal, vertical, lateral and yawing motions, though these conditions change in transition and forward flight. The dynamic equations to be decoupled are rewritten in state variable form in Eq. Al-8. The broken lines make $4 \times 4$ submatrices which clarify mann diagonal and off diagonal terms. The uncoupled equations with the no1se terms added become as given in Eq. A1-9.

$$
\left[\begin{array}{c}
B_{1 s} \\
-a \\
\delta_{1} \\
A_{1 s} \\
- \\
-
\end{array}\right]
$$

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In near hover flight one often assumes $u_{0}=0, \gamma_{0}=0$, and the following stability derivatives are often neglibly small and often assumed to be zero or:

$$
\begin{aligned}
& X_{W}^{\prime}=M_{W}=Z_{u}=0 ; X_{\delta_{c}}=M_{\delta_{c}}=Z_{B_{I S}}=0 \\
& N_{p}^{\prime}=N_{V}^{\prime}=L_{r}^{\prime}=0 ; Y_{\delta_{r r}}=L_{\delta_{r}}=N_{A_{I S}}^{\prime}=0
\end{aligned}
$$

With these assumption Equation Al-9 then becomes:
(continued next page)

$$
\begin{aligned}
& {\left[\begin{array}{c:cc}
-x_{u} & 0 & 0 \\
0 & 0 & 0 \\
& M_{u} & 0 \\
u_{0} & 0 \\
\hdashline 0 & Z_{w} & 0 \\
0 & 0 & -Y_{v} \\
0 & 0 & 0 \\
0 & 0 & -I_{v} \\
0 & 0 & -N_{v}^{-}
\end{array}\right]}
\end{aligned}
$$

## Helıcopter Transfer Functions

For conventional analysis, the equations of motions are often written in the Laplace domain. The longitudinal and lateral uncoupled equations of motion for the control rate derivatives and gust inputs are:

Longitudinal-Vertical Equations of Motion
Assumptions

$$
\begin{align*}
& X_{\dot{W}}=Z_{\dot{W}}=0, \theta_{0}=0, \\
& X_{\alpha}=Z_{\alpha}=M_{\alpha}=M_{\dot{\alpha}}=0 \\
& {\left[\begin{array}{cc:c}
s-x_{u} & -X_{w} & -X_{q} s+g \cos \gamma_{0} \\
-Z_{u} & s-Z_{w} & -\left(Z_{q}+U_{0}\right) s+g \sin \gamma_{0} \\
-M_{u} & -M_{w} s-M_{w} & s^{2}-M_{q}
\end{array}\right]\left[\begin{array}{l}
u \\
w \\
\theta
\end{array}\right]=\left[\begin{array}{cc}
x_{B_{1}} s+x_{B_{1}} & x_{\delta_{c}} \\
Z_{B_{1}}^{s+Z_{B_{1}}} & Z_{\delta_{c}} \\
M_{B_{1}} s+M_{B_{1}} & M_{\delta_{c}}
\end{array}\right]\left[\begin{array}{l}
B_{1 s} \\
\delta_{c}
\end{array}\right]+} \\
& {\left[\begin{array}{c:c}
-x_{u} & -x_{w} \\
-z_{u} & -z_{w} \\
-M_{u} & -M_{w}
\end{array}\right]\left[\begin{array}{l}
u_{g} \\
w_{g}
\end{array}\right]} \tag{AI-11a}
\end{align*}
$$

Lateral-Directional Equatıons of Motion
Assumptions

$$
Y_{\dot{V}}=L_{\dot{V}}=N_{\dot{v}}=0
$$

$$
\begin{align*}
& +\left[\begin{array}{c}
-Y_{v} \\
-\mathrm{L}_{v}^{-} \\
-\mathrm{N}_{v}^{-}
\end{array}\right]\left[\begin{array}{l}
\mathrm{v}_{\mathrm{g}}
\end{array}\right] \tag{Al-11b}
\end{align*}
$$

The basic equations of motion for a helicopter near hover with $\gamma=0$; $U_{0}=0$

Longitudinal:

$$
\begin{align*}
& {\left[\begin{array}{ccc}
s-X_{u} & -X_{w} & x_{q} s+g \\
-Z_{u} & s-Z_{w} & -Z_{q} s \\
-M_{u} & -M_{w} & s^{2}-M_{q} s
\end{array}\right]\left[\begin{array}{c}
u \\
w \\
\theta
\end{array}\right]\left[\begin{array}{lll}
x_{\dot{B}_{1}} & +x_{B_{1}} & x_{\delta_{c}} \\
z_{\dot{B}_{1}} & +Z_{B_{1}} & z_{\delta_{c}} \\
M_{\dot{B}_{1}} & +M_{B_{1 s}} & M_{\delta_{c}}
\end{array}\right]\left[\begin{array}{c}
B_{1} s \\
\delta_{c}
\end{array}\right]+} \\
& +\left[\begin{array}{cc}
-x_{u} & -x_{u} \\
-z_{u} & -Z_{w} \\
-M_{u} & -M_{w}
\end{array}\right]\left[\begin{array}{l}
u_{g} \\
u_{g} \\
w_{g} \\
g
\end{array}\right] \tag{A1-12a}
\end{align*}
$$

Lateral:

$$
\left[\begin{array}{lll}
s-Y_{v} & -Y_{p} s-g & -Y_{r} s  \tag{Al-12b}\\
-L_{v} & s^{2}-L_{p} s & -L_{r}^{\prime} s \\
-N_{v}^{\prime} & -N_{p}^{\prime} s & s^{2}-N_{r}^{\prime} s
\end{array}\right]\left[\begin{array}{l}
v \\
\phi \\
\psi
\end{array}\right]=\left[\begin{array}{ll}
Y_{A_{1 s}} & Y_{\delta_{r}} \\
L_{A_{1 s}} & L_{\delta_{r}} \\
N_{A} & N_{\delta_{r}}
\end{array}\right] \quad\left[\begin{array}{l}
A_{1} \\
\delta_{r}
\end{array}\right]+\left[\begin{array}{l}
-Y_{v} \\
-L_{v} \\
-N_{V}
\end{array}\right]\left[\begin{array}{l}
v_{g}
\end{array}\right]
$$

Often are neglıgible, small and neglected:

$$
\begin{array}{ll}
X_{W}=M_{W}=Z_{u}=0 & X_{\delta_{c}}=M_{\delta}=Z_{B_{I S}}=0 \\
N_{p}^{\prime}=N_{V}^{\prime}=L_{r}^{\prime}=0 & Y_{\delta_{c}}=L_{\delta_{r}}=N_{A_{1 s}^{\prime}}^{\prime}=0
\end{array}
$$

Also are often neglected the control rate terms

$$
\mathrm{X}_{\mathrm{B}_{1}}^{-}=\mathrm{Z}_{\mathrm{B}_{1}}=\mathrm{M}_{\dot{B}_{1}}=0 \quad \text { and } \quad \mathrm{X}_{\mathrm{q}}=\mathrm{Z}_{\mathrm{q}}=0
$$

The analog computer diagram of the longitudinal and lateral equations of motions are shown in Illustration A1-1 and A1-2.


Illustration A-1a. Lateral Equations


Illustration A-1b. Longitudinal Equation

Input-output Relationships (Hover)
The uncoupled sets of equations can be used to determine the inputoutput relationships and the effects of gust inputs.

The longitudinal equations are:

$$
\left[\begin{array}{c}
\dot{u}  \tag{A1-13}\\
\dot{\theta} \\
\dot{q}
\end{array}\right]=\left[\begin{array}{ccc}
+x_{u} & -g & x_{q} \\
0 & 0 & 1 \\
M_{u} & 0 & M_{q}
\end{array}\right]\left[\begin{array}{c}
u \\
\theta \\
q
\end{array}\right]+\left[\begin{array}{c}
x_{B_{1 s}} \\
0 \\
M_{B_{1 s}}
\end{array}\right]{B_{1 s}}+\left[\begin{array}{c}
-x_{u} \\
0 \\
-M_{u}
\end{array}\right] u_{g}
$$

The basic equations are:

$$
\begin{align*}
& \ddot{x}+X_{u} \dot{z}-g \theta+M_{q} \dot{\theta}=X_{B_{1}} B_{1 s}-X_{u} u_{g} \\
& M_{u} \dot{x}+\ddot{\theta}+M_{q} \dot{\theta}=M_{B_{1 s}} B_{1 s}-M_{u} u_{g} \tag{A1-14}
\end{align*}
$$

or

$$
\left[\begin{array}{cc}
s-X_{u} & x_{q} s+g  \tag{Al-15}\\
-M_{u} & s^{2}-M_{q} s
\end{array}\right]\left[\begin{array}{l}
u \\
\theta
\end{array}\right]\left[\begin{array}{l}
x_{B_{1 s}} \\
M_{B_{1}}
\end{array}\right]{ }^{B_{1 s}}+\left[\begin{array}{c}
-X_{u} \\
-M_{u}
\end{array}\right]{ }^{u_{g}}
$$

The characteristic equation is:

$$
\begin{align*}
\Delta & =s\left(s-M_{q}\right)\left(s-X_{u}\right)+M_{u} X_{q} s+g M_{u}  \tag{A1-16}\\
& =s^{3}-\left(M_{q}+X_{u}\right) s^{2}+\left(X_{u q} M_{q}+M_{u} X_{g}\right) s+g M_{u} \\
N_{B_{1 s}}^{u} & =\left|\begin{array}{ll}
X_{B_{1 s}} & X_{q} s+g \\
M_{B_{1 s}} & s\left(s-M_{q}\right)
\end{array}\right|
\end{align*}
$$

$$
\begin{align*}
N_{B_{1 s}}^{\theta} & =\left|\begin{array}{cc}
s-X_{u} & X_{B_{1}} \\
-M_{u} & M_{B_{1 s}}
\end{array}\right|  \tag{A1-18}\\
& =M_{B_{1 s}}\left(s-X_{u}+\frac{X_{B_{1 s}}}{\left.M_{B_{1 s}} M_{u}\right)}\right.
\end{align*}
$$

The Input-Output Relationships are:

$$
\begin{align*}
& \frac{U}{B_{1 s}}=\frac{X_{B_{1 s}}\left(s^{2}-\left(M_{q}-\frac{{ }_{B_{1 s}} X_{q}}{X_{B_{1}}}\right) s-\frac{M_{B_{1 s}}}{X_{B_{1 s}}} g\right)}{s^{3}-\left(X_{u}+M_{q}\right) s^{2}+\left(X_{u} M_{q}-M_{u} X_{q}\right) s+M_{u} g}  \tag{A1-19}\\
& \frac{X}{B_{1 s}}=\frac{1}{s} \cdot \frac{u}{B_{1 s}}  \tag{A1-20}\\
& \frac{\theta}{B_{1 s}}=\frac{M_{B_{1 s}}\left(s-X_{u}+\frac{X_{B_{1 s}}}{M_{B_{1 s}}} M_{u}\right)}{s^{3}-s^{2}\left(X_{u}+M_{q}\right)+\left(X_{u} M_{q}-M_{u} X_{q}\right) s+M_{u} g}  \tag{A1-21}\\
& \frac{Q}{B_{1 s}}=\frac{M_{B_{1 s}} s\left(s-X_{u}+\frac{X_{B_{1 s}}}{M_{B_{1 s}}} M_{u}\right)}{s^{3}-s^{2}\left(X_{u}+M_{q}\right)+\left(X_{u} N_{q}-M_{u} X_{q}\right) s+M_{u} g}  \tag{A1-22}\\
& =\frac{s \theta}{B_{1 s}}
\end{align*}
$$

$$
\begin{align*}
\frac{u}{u_{g}}= & \frac{-X_{u}\left(s^{2}-\left(M_{q}-\frac{M_{u} X_{q}}{X_{u}}\right) s-\frac{M_{u}}{X_{u}} g\right)}{s^{3}-s^{2}\left(X_{u}+M_{q}\right)+\left(X_{u} M_{q}-M_{u} X_{q}\right) s+M_{u} g}  \tag{A1-23}\\
\frac{x}{u_{g}}= & \frac{-X_{u}\left(s^{2}-\left(M_{q}-\frac{M_{u} X_{q}}{X_{u}}\right) s-\frac{M_{u}}{X_{u}} g\right)}{s\left[s^{3}-s^{2}\left(X_{u}+M_{q}\right)+\left(X_{u} M_{q}-M_{u} X_{q}\right) s+M_{u} g\right]} \tag{A1-24}
\end{align*}
$$

The lateral equations are:

$$
\left[\begin{array}{c}
\dot{v}  \tag{A1-25}\\
\dot{\phi} \\
\dot{p}
\end{array}\right]=\left[\begin{array}{ccc}
Y_{v} & g & Y_{p} \\
0 & 0 & I \\
L_{v} & 0 & L_{p}
\end{array}\right]\left[\begin{array}{c}
v \\
\phi \\
p
\end{array}\right]+\left[\begin{array}{c}
Y_{A_{1}} \\
0 \\
L_{A_{1 S}}
\end{array}\right] A_{1 s}+\left[\begin{array}{c}
-Y_{v} \\
0 \\
-\mathrm{L}_{v}
\end{array}\right] \mathrm{v}_{\mathrm{v}}
$$

The basic equations are:

$$
\begin{align*}
& \dot{\mathrm{V}}=Y_{v} v+g \phi+Y_{p} p+Y_{A_{1 s}} A_{1 s}  \tag{Al-26}\\
& \dot{\phi}=L_{v} v+L_{p} p+L_{A_{1 s}} A_{1 s}
\end{align*}
$$

or

$$
\left[\begin{array}{cc}
s-Y_{v} & -\left(g+Y_{p} s\right)  \tag{A1-27}\\
-L_{v} & s\left(s-L_{p}\right)
\end{array}\right]\left[\begin{array}{l}
v \\
\Phi
\end{array}\right]=\left[\begin{array}{c}
Y_{A_{1 s}} \\
L_{A_{1 s}}
\end{array}\right] A_{1 s}+\left[\begin{array}{c}
-Y_{v} \\
-L_{v}
\end{array}\right] \quad v_{g}
$$

The characteristic equation is:

$$
\begin{align*}
& \Delta=\left|\begin{array}{lr}
s-Y_{v} & -\left(g^{+} Y_{p} s-\right) \\
-L_{v} & s\left(s-L_{p}\right)
\end{array}\right|  \tag{Al-28}\\
& =s^{3}-\left(L_{p}+Y_{v}\right) s^{2}+\left(L_{p} Y_{v}-L_{v} Y_{p}\right) s-L_{v} g \\
& N_{A_{1 s}}^{V}=\left|\begin{array}{ll}
Y_{A_{1 s}} & -\left(g+Y_{p} s\right) \\
L_{A_{1 s}} & s\left(s-L_{p}\right)
\end{array}\right|  \tag{A1-29}\\
& =Y_{A_{1 s}}\left[s^{2}-\left(L_{p}-\frac{L_{A}}{Y_{A_{1 s}}} Y_{p}\right) s+\frac{L_{A} 1 s}{Y_{A_{1 s}}} g\right] \\
& N_{A_{1 s}}^{\phi}=\left|\begin{array}{ll}
s-Y & Y_{A_{1 s}} \\
-L_{v} & L_{A_{1 s}}
\end{array}\right|  \tag{A1-30}\\
& =L_{A_{1 s}}\left[s-Y_{V}+\frac{A_{1 s}}{L_{A_{1 s}}} L_{v}\right] \\
& N_{v_{g}}^{v}=\left|\begin{array}{ll}
-Y_{v} & -\left(g+Y_{p} s\right) \\
-L_{v} & s\left(s-L_{p}\right)
\end{array}\right|  \tag{A1-31}\\
& =-Y_{v}\left(s^{2}-\left(L_{p}-\frac{L_{v}}{Y_{v}} Y_{p}\right) s+\frac{L_{v}}{Y_{v}} g\right)
\end{align*}
$$

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$$
\begin{align*}
N_{v}^{\phi} & =\left|\begin{array}{cc}
s-Y_{v} & -Y_{v} \\
-L_{v} & -L_{v}
\end{array}\right|  \tag{A1-32}\\
& =-L_{v}\left(s-Y_{v}+Y_{v}\right)=-L_{v} s
\end{align*}
$$

The input-output relationships are:

$$
\begin{align*}
& \frac{Y}{A_{1 s}}=\frac{Y_{A_{1 s}}\left[s^{2}-\left(L_{p}-\frac{L_{A}}{Y_{A}} Y_{p}\right) s+\frac{L_{A_{1 s}}}{Y_{A_{1 s}}} g\right]}{s\left[s^{3}-\left(L_{p}+Y_{v}\right) s^{2}+\left(L_{p} Y_{v}-L_{v} Y_{p}\right) s-L_{v} g\right]}  \tag{A1-33}\\
& \frac{Y}{A_{1 s}}=\frac{Y_{A_{1 s}}\left[s^{2}-\left(L_{p}-\frac{L_{A}}{I_{1 s}} Y_{p}\right) s+\frac{L_{A_{1 s}}}{Y_{1 s}} g\right]}{\left[s^{3}-\left(L_{p}+Y_{v}\right) s^{2}+\left(L_{p} Y_{v}-L_{v} Y_{p}\right) s-L_{v} g\right]}  \tag{A1-34}\\
& \frac{\phi}{A_{1 s}}=\frac{L_{A_{1 s}}\left(s-Y_{v}+\frac{Y_{A_{1 s}}}{L_{A}} L_{v}\right)}{s^{3}-\left(L_{p}+Y_{v}\right) s^{2}+\left(L_{p} Y_{v}-L_{v} Y_{p}\right) s-L_{v} g}  \tag{A1-35}\\
& \frac{\dot{\phi}}{A_{I s}}=\frac{L_{A_{1 s}} s\left(s-Y_{u}+\frac{Y_{A_{1 s}}}{L_{A}} L_{u}\right)}{s^{3}-\left(L_{p}+Y_{v}\right) s^{2}+\left(L_{p} Y_{v}-L_{v} Y_{p}\right) s-L_{v} g}  \tag{A1-36}\\
& \frac{X}{V_{g}}=\frac{-Y_{v}\left[s^{2}-\left(L_{p}-\frac{L_{v}}{Y_{v}} Y_{p}\right) s+\frac{L_{v}}{Y_{V}} g\right.}{s\left[s^{3}-\left(L_{p}+Y_{u}\right) s^{2}+\left(L_{p} Y_{v}-L_{v} Y_{p}\right) s-L_{v} g\right]} \tag{Al-37}
\end{align*}
$$

$$
\begin{align*}
& \frac{v}{V}=\frac{-Y_{v}\left(s^{2}-\left(L_{p}-\frac{L_{v}}{Y_{u}} Y_{p}\right) s+\frac{L_{v}}{Y_{u}} g\right)}{s^{3}-\left(L_{p}+Y_{v}\right) s^{2}+\left(L_{p} Y_{v}-L_{v} Y_{p}\right) s-L_{v} g}  \tag{Al-38}\\
& \frac{\Phi}{\frac{V}{V}}=\frac{-L_{v} s}{s^{3}-\left(L_{p}+Y_{v}\right) s^{2}+\left(L_{p} Y_{v}-L_{v} Y_{p}\right) s-L_{v} g}  \tag{Al-39}\\
& \frac{\Phi}{V_{g}}=\frac{-L_{v} s^{2}}{s^{3}-\left(L_{p}+Y_{v}\right) s^{2}+\left(L_{p} Y_{v}-L_{v} Y_{p}\right) s-L_{v} g} \tag{A1-40}
\end{align*}
$$

The vertical equation of motion is:

$$
\begin{equation*}
\ddot{w}=Z_{w} w+Z_{\delta_{e}} \delta_{c}-Z_{w} w_{g} \tag{A1-41}
\end{equation*}
$$

The input-output relationships are:

$$
\begin{align*}
& \frac{W}{\delta_{c}}=\frac{Z_{\delta_{c}}}{s-Z_{w}} \hat{\delta}_{c}  \tag{A1-42}\\
& \frac{Z}{\delta_{c}}=\frac{Z_{\delta_{c}}}{s\left(s-Z_{w}\right)} \delta_{c}  \tag{Al-43}\\
& \frac{W}{W_{g}}=\frac{-Z_{W}}{s-Z_{W}} W_{g} \tag{Al-44}
\end{align*}
$$

$$
\frac{z}{W_{g}}=\frac{-X_{w}}{s\left(s-X_{w}\right)} w_{g} \quad \begin{align*}
& \text { ORIGINAL PAGE IS }  \tag{A1-45}\\
& \text { OF POOR QUALITY }
\end{align*}
$$

The yawng equation of motion is:

$$
\begin{equation*}
\dot{r}=N_{r} r+N_{\delta_{r}} \delta_{r} \tag{A1-46}
\end{equation*}
$$

The input-output relationships are:

$$
\begin{align*}
& \frac{R}{\delta_{r}}=\frac{{ }^{N_{\delta_{T}}}}{S-N_{r}}  \tag{Al-47}\\
& \frac{\Psi}{\delta_{r}}=\frac{N_{\delta_{r}}}{s\left(s-N_{r}\right)} \tag{A1-48}
\end{align*}
$$

REFERENCES

A1 Curtiss, H.C., Jr., "Some Notes on VTOL Stability and Contro1", Princeton University Course Notes, August 1971.

A2 McRuer, D., Ashkenas, I., \& Graham, D., "Aircraft Dynamıcs and Automatic Control", System Technology Inc., Hawthorne, California (to be published).

In this table the numerical data used in the example is listed. The data is comparable to a heavy lift hellcopter CH54B gross weight 47000 lbs.

The inertia are:

$$
M=1460 \text { slugs }
$$

$$
I_{x x}=47,020 \text { slug } \mathrm{ft}^{2}
$$

$$
I_{y y}=192,700 \text { slug } \mathrm{ft}^{2}
$$

$$
I_{z z}=164,100 \text { slug } \mathrm{ft}^{2}
$$

$$
I_{x z}=-19,570 \text { slug } \mathrm{ft}^{2}
$$

The full control range is:
$\delta_{c}(41.0) \quad .3 \mathrm{rad}$
$A_{\text {Is }}$ (40) . 28 rad
$\mathrm{B}_{1 \mathrm{~s}}$ (28.5 . 455 rad
$\delta_{t}(13.6) .54 \mathrm{rad} \mathrm{w} / \mathrm{o} . \mathrm{t}$. with over travel T.R.

The limitations on the controls are (in degrees):
Cyclic $26^{\circ}\left\{^{-6^{\circ}}+19^{\circ}\right.$
Lateral $16^{\circ}\left\{^{-2.5^{\circ}}+13.5^{\circ}\right.$
Collective $6.6^{\circ}-23.8^{\circ}$
Directional $31^{\circ}$ ( $20^{\circ}$ without over travel)

Table A2. Numerical Values CH54B Hover O.G.E.
The dimensional numerical values are normallzed by the inertia G.W. $=47,000 \mathrm{lbs}$.

|  | U | V | W | $\mathrm{p}=\dot{\phi}$ | $\mathrm{q}=\dot{\theta}$ | $\mathrm{r}=\dot{\psi}$ |
| :--- | :---: | :---: | :---: | :---: | :---: | :---: |
| X | -.0169 | -.0008 | .0057 | -2.3082 | .6612 | -.2853 |
| Y | .0016 | -.0405 | -.0160 | -.8945 | -2.1780 | 1.2315 |
| Z | .0044 | -.0150 | -.2689 | -.0244 | .2014 | 2.6315 |
| L | .0012 | -.0153 | -.0032 | -1.0937 | -1.2943 | .3496 |
| M | .0024 | .0003 | -.0008 | .3513 | -.3569 | -.0072 |
| N | -.0002 | .0088 | .0028 | -.0147 | .0750 | -.5199 |


|  | $\delta_{c}$ | $\mathrm{~A}_{1 \mathrm{~s}}$ | $\mathrm{~B}_{1 \mathrm{~s}}$ | $\delta_{t}$ |
| :---: | :---: | ---: | :---: | :---: |
| X | 6.1041 | -1.2890 | +35.6438 | -.0159 |
| Y | -18.0959 | +36.1232 | +1.1884 | 23.952 |
| Z | -292.260 | -2.2376 | 1.0596 | -.0339 |
| L | -5.387 | 23.4155 | -.1215 | 6.5737 |
| M | -.1719 | .0648 | -5.6617 | -.1756 |
| N | 11.627 | .0521 | .2236 | 9.5612 |

Table A3. Numerical Values CH54B Airspeed 60 Knots

The dimensional numerical values are nomalized by the inertia G.W. - 47,000 Ibs.

|  | U | V | W | $\mathrm{p}=\dot{\phi}$ | $\mathrm{q}=\dot{\theta}$ | $\mathrm{r}=\dot{\psi}$ |
| :--- | :---: | :---: | :---: | :---: | :---: | :---: |
| X | -.0221 | -.0028 | .0049 | -2.073 | 1.515 | -.2431 |
| Y | .0027 | -.0717 | -.0286 | -1.7885 | -2.024 | 1.990 |
| Z | -.100 | -.030 | -.5533 | -1.3915 | 1.0951 | 2.1848 |
| L | .0016 | -.0168 | -.0046 | -1.402 | -1.2729 | .5149 |
| M | .0020 | -.0003 | -.0016 | .2814 | -.4254 | .0228 |
| N | -.003 | .0144 | -.0002 | .0297 | .2506 | -.8294 |


|  | $\delta_{\mathrm{c}}$ | $\mathrm{A}_{1 \mathrm{~s}}$ | $\mathrm{~B}_{1 \mathrm{~s}}$ | $\delta_{\mathrm{t}}$ |
| :--- | :---: | :---: | :---: | :---: |
| X | -5.4716 | -2.4373 | 33.1214 | -.3398 |
| Y | -15.229 | 35.353 | 4.720 | 20.842 |
| Z | -304.04 | -1.440 | 57.749 | .3203 |
| L | -2.2480 | 22.82 | .9092 | 5.7422 |
| M | 1.2833 | .2814 | -5.355 | .0710 |
| N | 7.6112 | -.1179 | .224 | -8.3729 |

## APPENDIX B1

DECOUPLING METHOD (LEAST SQUARES)

## Decoupling Using the Least Squares Method

When the number of outputs of interest exceed that of control inputs, the least squares method is employed by minimizing values of coupling terms. Consider the following algebraic matrix equation.

$$
\begin{equation*}
\dot{A} X=B \tag{Bl-1}
\end{equation*}
$$

where

$$
\begin{aligned}
& \text { A : n x m matrix } \\
& \text { B : n x } 1 \text { matrix } \\
& x: m \times 1 \text { unknown matrix }(m<n)
\end{aligned}
$$

An approximate solution is obtained by minimizing the following:

$$
\begin{equation*}
\text { Minimize }\lfloor|A X-B| \mid \tag{Bl-2}
\end{equation*}
$$

where $\|\cdot\|$ stands for an enclidean norm given by ${ }_{1}$

$$
\begin{equation*}
\left\lfloor\lfloor A X-B\rfloor \left\lvert\,=\left\{\operatorname{Trace}\left[(A X-B)(A X-B)^{T}\right]^{-\frac{1}{2}}\right.\right.\right. \tag{Bl-3}
\end{equation*}
$$

Then, the solution becomes:

$$
\begin{equation*}
X=\left(A^{T} A\right)^{-1} B^{T} B \tag{Bl-4}
\end{equation*}
$$

And

$$
\begin{equation*}
A X=A\left(A^{T} A\right)^{-1} B^{T} B \cong B \tag{Bl-5}
\end{equation*}
$$

The proof is as follows:
The solution matrix $X$ is given by:

$$
\begin{equation*}
\frac{\partial}{\partial x}\lfloor A X-B \|=0 \tag{BI-6}
\end{equation*}
$$

$$
\begin{array}{ll}
\therefore & \frac{\partial}{\partial X}\left\{\operatorname{Trace}\left[(A X-B)(A X-B)^{T}\right]\right\}^{\frac{1}{2}}=0 \\
\therefore & A^{T}(A X-B)=0 \tag{Bl-8}
\end{array}
$$

$\therefore \quad X=\left(A^{T} A\right)^{-1} A^{T} B$, if $A$ has no same column.
This proof uses the following properties of the Trace:

$$
\begin{align*}
& \frac{\partial}{\partial x} \operatorname{Trace}[A x]=A^{T}  \tag{Bl-10}\\
& \frac{\partial}{\partial x} \operatorname{Trace}\left[x x^{T}\right]=2 X \tag{B1-11}
\end{align*}
$$

A weighting for rows of the matrix equation (B3-1) can be used to increase the relative accuracy of approximation. The norm becomes:

$$
\begin{equation*}
\|Q(A X-B)\|=\left\{\operatorname{Trace}\left[(A X-B)^{T} Q^{T} Q(A X-B)\right]\right\}^{\frac{1}{2}} \tag{Bl-12}
\end{equation*}
$$

where the weighting matrix $Q$ is given by:


The solution is:

$$
\begin{equation*}
X=\left(A^{T} \underset{\sim}{Q}\right)^{-1} A Q_{3} \tag{Bl-14}
\end{equation*}
$$

where

$$
\begin{equation*}
Q=Q^{T} Q \tag{Bl-15}
\end{equation*}
$$

The limitations of this method can be illustrated as follows: The dynamic equation of the system is:

$$
\begin{aligned}
& x=A x+B u \\
& y=x \\
&\text { (A) If } \left.B_{i} \text { ( } i=\text { th }{ }^{\prime} \text { row }\right)=0 \text {, the corresponding i-th state equation } \\
&(B 1-17)
\end{aligned}
$$ should not include coupling terms.

(B) If $y=p x$, the output matrix $P$ has only one non-zero element In the same column so that desirable state matrix $A^{*}$ and control matrix B* can be defined.

## DECOUPLING OF HELICOPTER DYNAMICS

Decoupling in the multivariable systems has been studied by many authors. Morgan (Ref. BI) first posed the decoupling problem. Falb and Volvoich (Ref. B2) gave necessary and sufficient condition for decoupling. Gilbert and some authors (Ref. B3, B4, and B5) extended this method. However, these methods are stall cumbersome in algebraic manipulation and are often not applicable to control problems where the number of controls is less than the number of control system outputs.

In this work is used a decoupling method of least squares.
The dynamical equations are given by:

$$
\begin{align*}
& \dot{x}=A x+B u  \tag{B2-1}\\
& y=x  \tag{B2-2}\\
& u=C k x+C \delta \tag{B2-3}
\end{align*}
$$

where

```
x : n - state vector
y : n - output vector
u : m - control vector
\delta :m - control input vector
A : n x n system matrix
B : n x m control matrix
C : m x m control decoupling matrix
K : m x n decoupling state feedback matrıx
```

Substituting Eq. B2-3 into B2-1 and obtain the following equation:

$$
\begin{align*}
& \dot{x}=(A+B C K 1 x+B C \delta  \tag{B2-4}\\
& y=x \tag{B2-5}
\end{align*}
$$

Then, the block diagram for decoupling is shown in Illustration B2-1.


Illustration B2-1. Block Diagram for Decoupling

When the desirable decoupled control matrices are respectively $\mathrm{B}^{*}$ and $A^{+}$, then the following relations are satisfied:

$$
\begin{align*}
\mathrm{BC} & =\mathrm{B}^{*}  \tag{B2-6}\\
\mathrm{~A}+\mathrm{BCK} & =\mathrm{A}^{*}
\end{align*}
$$

From these equations, using a pseudo-inverse matrix which is a solution of least squares method, as shown in Appendix BI, one obtains the decoupling matrices $C$ and $K$.

$$
\begin{align*}
C & =\left(B^{T} B\right)^{-1} B^{T} B^{*}  \tag{B2-8}\\
K & =\left(C^{T} B^{T} B C\right)^{-1} C_{-}^{T} B^{T}\left(A-A^{*}\right)  \tag{B2-9}\\
& \doteqdot\left(B^{*} T_{B}\right)^{-1} B^{* T}\left(A-A^{*}\right)
\end{align*}
$$

Now, the decoupling matrices $(C, K)$ decouple the system completely or approximately. The practical example of the almost decoupled system is shown in Illustration $B 2-1$. This method is used for decoupling of the helicopter dynamics at hover and approach phase.

Decoupling the helicopter (at hover)
The dynamic equation is rewritten, using a numerical hover example, as follows: (eg. 2.8) with cancelled inertia coupling term)

And the desirable control matrix $B^{*}$ and the desirable system matrix $A^{*}$ are assumed to be given by Eq. ( $\mathrm{B} 2-\mathrm{EX}$ ) and Eq. (B2-12).

$$
B^{*}=\left[\begin{array}{c:c:c:c}
35.6 & 0.0 & 0.0 & 0.0  \tag{B2-11}\\
0 & : & 0 & 0 \\
0 & 0 & 0 \\
\hdashline-5.66 & 0.0 & 0.0 & 0.0 \\
\hdashline 0.0 & -292 . & 0.0 & 0.0 \\
\hdashline 0.0 & 0.0 & 36.1 & 0.0 \\
0 & 0 & 0 & 0 \\
0.0 & 0.0 & 24.6 & 0 \\
\hdashline 0.0 & 0.0 & 0.0 & -10.9
\end{array}\right]
$$



These matrices $\mathrm{B}^{*}$ and $\mathrm{A}^{*}$ are chosen in such a way that characterıstics of the decoupled helicopter are similar to those of the basic helicopter. After being decoupled, the characteristics would be improved by the conventional technique or the optimal control technique.

After simple calculations following Eq. (B2-8) and (Eq. B2-9), one obtains the decoupling matrices C and K given by Eq . ( $\mathrm{B} 2-13$ ) and Eq. (B2-14).

$$
C=\left[\begin{array}{rrrr}
.998 & -.176 & .042 & -.025  \tag{B2-13}\\
.004 & 1.000 & -.009 & .004 \\
-.014 & -.159 & 1.152 & -.576 \\
-.005 & 1.085 & -.266 & .977
\end{array}\right]
$$

The member of the diagonal submatrices of the matrix $K$ are about a factor $10^{-1}$ less than those of the same column and rows. This means that the feedback gains of the main diagonal submatrices, which artificially govern characteristics of the helicopter dynamics, have no effect on decoupling with exception of the yawing motion. However, the latter is sufficiently stable as is apparent from Eq. (B2-14). This indicates that there exist free choice of main diagonal feedback gains without deficiency in decoupling.

Substituting Eq. (B2-14), (B2-13) into Eq. (B2-4), gives the almost decoupled helicopter dynamics as follows:


In order to study the degree of decoupling for a control input, an analogue simulation was done as shown in Figures ( $\mathrm{B} 2-1$ ) to ( $\mathrm{B} 2-8$ ). The decoupling reduces the longitudinal/lateral coupling terms by a factor of 10 or more. The coupling between the vertical mode and other modes also decreases, showing significant reduction between the vertical and yawing mode. The yawing response is improved for the control input, while its coupling with the other axises decreases significantly. A change in some control derivatives signs accompany turning direction of initial motion opposite to that of basic derivatives. An evaluation of this sign change may require a simulator test by a pilot in the case of anstable helicopter.

Decoupling the Helicopter (at approach phase)
The dynamic equation, using the numerical example for the approach plane, becomes as follows: (with cancelled inertia coupling term)
and the desirable control matrix $B^{*}$ and the desirable system matrix A* are assumed to be g1ven by Eq. (B2-18) and Eq. (B2-19).

$$
B^{*}=\left[\begin{array}{cccc}
33.12 & 0.0 & 0.0 & 0.0  \tag{B2-18}\\
0 & 0 & 0 & 0 \\
-5.35 & 0.0 & 0.0 & 0.0 \\
\hdashline-0 & -304.4 & 0.0 & 0.0 \\
\hdashline 0.0 & 0.0 & 35.35 & 0.0 \\
0 & 0 & 0 & 0 \\
0.0 & 0.0 & 24.06 & 0 \\
\hdashline 0.0 & 0.0 & 0.0 & -9.53
\end{array}\right]
$$


(B2-18)
These matrices $\mathrm{B}^{*}$ and $\mathrm{A}^{*}$ are chosen in such a way that characteristics of the decoupled helicopter are similar to those of the basic helicopter. After being decoupled, the characteristics would be improved by the conventional technique or the optimal control technique.

After simple calculations following Eq. (B2-16) and Eq. (B2-17), one obtains the decoupling matrices C and K given by Eq. (B2-21) and-Eq. (B2-22).
$C=\left[\begin{array}{cccc}1.02 & .17 & .08 & -.034 \\ .19 & 1.03 & .010 & .003 \\ -.1 & -.14 & 1.15 & -.5 \\ .13 & .95 & -.30 & .986\end{array}\right]$
(B2-19)

Substituting Equation B2-9 and B2-20 into Equation B2-5, gives the almost decoupled helicopter dynamics as follows.

(B2-21)
$\left[\begin{array}{cccc}33.02 & .077 & .041 & -.210 \\ 0 & 0 & 0 & 0 \\ -5.24 & .43 & -.137 & .106 \\ -.0008 & -304.04 & .0003 & -.0018 \\ .01 & .077 & 34.76 & 2.10 \\ 0 & 0 & 0 & 0 \\ 1.39 & .11 & 24.8 & -.029 \\ .79 & -.057 & -.459 & -7.88\end{array}\right]\left[\begin{array}{c}\mathrm{B}_{1 \mathrm{~s}} \\ \delta_{\mathrm{c}} \\ \mathrm{A}_{1 \mathrm{~s}} \\ \delta_{r}\end{array}\right]$

Note that in the "decoupled" equations, the coupling terms are an order of ten smaller, compared to those of basic equations.

In the numerical values of the above equations is used ( $\mathrm{Z}_{\mathrm{q}}-\mathrm{U}_{0}$ ) with $U_{0}=0$. If $Z_{q}-U_{0}$ is used with $U_{0}=60$ knots $/ \mathrm{hr}$, the feedback gains $k_{i 1}$ remain the same with the exception of $k_{34}$, which becomes equal to -100 i.e., directly proportional to speed.

## Evaluation of Decoupling

The root configuration for basic, decoupled and uncoupled simplified dynamics are shown in Figure 2. The roots of the decoupled longitudinal equations are very close to those of the uncoupled equations. However, decoupling of the lateral equations shifts one of the modes to the right hand side in the $s$ plane, resulting in an unstable mode.

In order to study the degree of decoupling for a control input, an analog simulation was done in Reference (32). The decoupling reduces considerably the longitudinal/lateral coupling terms. The coupling between the vertical mode and other modes also decreases, showing significant reduction between the vertical and yawing mode. The yawing response is imporved for the control input, while its coupling with the other axes decreases signnficantly. A change in some control derivatıves signs accompany turning direction of initial motion opposite to that of basic derivatives. An evaluation of this sign change may require a simulator test by a pilot in the case of an unstable helicopter.

## Consideration for Practical Design

In general, the stability derivatives and control derivatives are a function of air velocity, altitude and time. Therefore, each element of the decoupling matrices $[\mathrm{C}, \mathrm{K}]$ must be given by:

$$
\begin{align*}
& C_{i j}=C_{i j}(V, h, t)  \tag{B2-23}\\
& K_{i j}=K_{i j}(V, h, t)
\end{align*}
$$

However, the derivatives are determined at a given air velocity and altitude. In practice, the altitude and time dependence can often be neglected and the derivatives are only a function of air velocity or:

$$
\begin{align*}
& C_{i j}=C_{i j}(V) \\
& K_{i j}=K_{i j}(V) \tag{B2-24}
\end{align*}
$$

When the derivatives between different velocities can be approximated by straight lines or simple curves variations in $C_{i j}$ and $K_{i j}$ would be easily programmed.

The decoupling matrices $C$ and $K$ are constructed as shown in Dllustration B2-
3. The assumptions for the design of the helicopter controller are as follows:
(I) The state variables $u, v, w, \theta, q, \phi, p$ and $r$ can be measured by sensors or approximated.
(II) The fly-by wire system would be desirable.
(III) The measurement of air velocity $V$ is possible. $=$


## REFERENCES

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## SENSITIVITY ANALYSIS

Sensitivity of Characteristic Roots to Variation in the System
$\underline{\text { Coefficients }{ }^{\left[a_{i j}\right.}{ }^{1}}$
The system equation with control free is given by ( $61-1$ ).

$$
\begin{equation*}
\dot{x}(t)=A x(t) \tag{C1-1}
\end{equation*}
$$

where

$$
\begin{aligned}
x(t) & : n \text {-state vector } \\
\mathrm{A} & : \mathrm{n} \times \mathrm{n} \text { system matrix }
\end{aligned}
$$

the $n$-th order characteristic equation is given by ( $C 1-2$ ).

$$
\begin{equation*}
|\mathrm{sI}-A|=0 \tag{C1-2}
\end{equation*}
$$

The sensitivity is defined as

$$
\begin{equation*}
s_{i j}^{k}=a_{i j} \frac{\Delta r_{k}}{\Delta a_{i j}} \tag{C1-3}
\end{equation*}
$$

where,

$$
\begin{aligned}
\mathrm{s}_{\mathrm{ij}}^{\mathrm{k}}: & \text { sensitivity of } k \text {-th characteristic root } \\
& \text { to } a_{i j} \text { variation } \\
a_{i j}: & \text { non-zero }(i, j) \text { element of } A \\
\Delta a_{i j}: & \text { variation in } a_{i j} \\
\Delta r_{k}: & \text { k-th root of the characteristic equation }
\end{aligned}
$$

And, $S_{i j}^{k}$ is a vector since $\Delta r_{k}$ is represented by:

$$
\begin{equation*}
\Delta r_{k}=\left\|\Delta r_{k}\right\| \tan ^{-1} \frac{\operatorname{Im}\left[\Delta r_{k}\right]}{\operatorname{Re}\left[\Delta r_{k}\right]} \tag{CI=4}
\end{equation*}
$$

Also, the following equations are easily proved with respect to $S_{i j}^{k}$, since Trace [A] is the sum of all roots.

$$
\begin{align*}
& \sum_{k=1}^{n} S_{i j}^{k}=a_{i j} \quad i=j, i, j=1,2, \ldots \ldots, n  \tag{C1-5}\\
& \sum_{k=1}^{n} S_{i j}^{k}=0 \quad i \neq j, i, j=1,2, \ldots \ldots, n \tag{C1-6}
\end{align*}
$$

Furthermore, the poles and zeros of the root locus of $a_{i j}$ variation are given by roots of (C1-7) and (C1-8) respectively.

$$
\begin{gather*}
|s I-A|=0  \tag{C1-7}\\
\operatorname{cofactor}\left(-a_{i j}\right)=0 \tag{Cl-8}
\end{gather*}
$$

These are derived as follows:
Letting the variation of the system matrix, $\triangle A$, the characteristic equation is rewritten by:

$$
\begin{equation*}
|s I-A-\Delta A|=0 \tag{e1-9}
\end{equation*}
$$

This equation can be equivalent to (Cl-10)

$$
\begin{align*}
& \left|I-(s I-A)^{-1} \Delta A\right|=0  \tag{C1-10}\\
& \cdot\left|I-\frac{\operatorname{adj}(s I-A) \Delta A}{\operatorname{det}(s I-A)}\right|=0 \tag{C1-11}
\end{align*}
$$

$$
\begin{aligned}
& \text { (C1-12) }
\end{aligned}
$$

If one assumes that

$$
\begin{align*}
& \therefore\left|I-\frac{\operatorname{adj}(s I-A) \Delta A}{\operatorname{det}[s I-A]}\right|=1-\frac{\Delta a_{i j} \operatorname{cof}\left(-a_{i j}\right)}{\operatorname{det}[s I-A]} \tag{Cl-15}
\end{align*}
$$

That is, the characteristic equation is:

$$
\begin{equation*}
1-\frac{\Delta a_{i j} \operatorname{cof}\left(-a_{i j}\right)}{\operatorname{det}[s I-A]}=0 \tag{CD-16}
\end{equation*}
$$

The poles and zeros are given from (c1-16)

The contributions of stability derivatives to control and dynamic stability are shown in Figures ( $C-1$ ) to ( $C-36$ ), with mostly expanded sensitivities. By these figures we can appreciate the following:
(1) Which roots (or modes) are most sensitive to the specific stability derivative?
(2) Which roots (or modes) are more sensitive to which stability derivatives?
(3) Existence of zeroes on real axis can be known.
(4) From (1) and (2) we can infer degree of coupling.

With respect to $L_{r}$ and $N_{u}$, root locus are shown with reducing $L_{r}$ and $N_{u}$ in Figure ( $C-37)$ and Figure ( $\mathcal{C}-38$ ), in order to study reduction in stability due to decoupling.











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Optimal Time-Invariant Linear Regulator (Deterministıc)
```

The general results of optimal time-invariant linear regulator problems are summarized in this section.

System Dynamics
Given is a deterministic, time-invariant, linear dynamical system with $n$-dimensional state vector (or variable) $x(t)$ and m-dimensıonal control vector (or variable) $u(t)$.

$$
\begin{align*}
& \dot{x}(t)=A x(t)+B u(t)  \tag{D1-1}\\
& x\left(t_{0}\right)=x_{0} \tag{D1-2}
\end{align*}
$$

where $A$ and $B$ are $n x n$ and $n x$ constant coefficient matrices, and $x_{0}$ is the initial state vector at time $t_{o}$. Usually, the dimensional of state vector is larger than that of control vector ( $n>m$ ).

The p-dimensional output vector $y(t)$ consists of a linear combination of the components of state vector $x(t)$ :

$$
\begin{equation*}
y(t)=H x(t) \tag{01-3}
\end{equation*}
$$

Without loss of generality, the $p \times n$ output matrix $H$ has its maximum rank $p(p \leq n)$. The output vector $y(t)$ can be regarded as $p$ independent
measurements of the system state $x(t)$. Note that, in many or most of the practical cases, the output matrix $H$ can be an $n x n$ identaty matrix. In IIlustration ( $D 1-1$ ) is shown the block diagram of the system.

Throughout this work, the lower case character usually stands for a column vector and the upper case for a matrix. The argument "t" usually means time and (•) represents the time derivative of the argument.

## Cost Function

By properly chosen output weighting matrix $Q$ and control weighting matrix $R$, the quadratic cost function without cross coupled term of the output vector and the control vector is expressed as follows

$$
J\left(x_{0}, u(t)\right)=\int_{t_{0}}^{\infty}\left[y^{T}(t) Q y(t)+u^{T}(t) R u(t)\right] d t
$$

where the matrices $Q$ and $R$ are $p x p$ and $m x m$ constant symmetric matrices, and throughout this report, the superscript $\left.T\left(C^{\cdot}\right)^{T}\right)$ stands for the transpose of vector or matrix.

The pxp symmetric output weighting matrix $Q$ and the mxm symmetric control weighting matrix $R$ are both assumed to be positive definite, or

$$
Q>0_{\}}
$$

Hence, the integrand of quadratic cost function $J\left(x_{0}, u(t)\right)$ in $E q$. ( $D i-4$ ) is non-negative for any output $y(t)$ and any control $u(t)$ during the process interval $\left[t_{0}, \infty\right)$.

## Controllability

An optimal control law $u^{*}(t)$ has to be determined, that minimizes the quadratic cost function $J\left(x_{0}, u(t)\right)$ subject to the system dynamics represented by Eqs. (D1-1), (D1-2) and (D1-3).

The assumption of the positive definiteness for $Q$ and $R$ weighting matrices guarantees the convexity of quadratic cost function $J_{\min }\left(x_{0}, u(t)\right)$, but it does not imply that the minimum of cost function $J_{m n}\left(x_{0}\right)$ is finite. The cost function is finnte by the "complete controllability condition".

The condition of "Complete Controllability" for stationary system (Eqs. (D1-1) and (1) $1-2$ ) considered here is that the following $n$ rows by nxnm composite matrix has the maximum rank $(=n)$ :

$$
\operatorname{Rank}\left[B: A B: A^{2} B: \ldots \ldots \ldots \ldots \ldots \ldots A^{n-1} A^{n}\right]=n \quad(D 1-6)
$$

A matrix pair of $A$ and $B$ satisfying Eq. (D1-6) is usually said as "Completely Controllable pair [A, B]'.

Complete controllability guarantees the finiteness of the minimum cost function $J_{\min }\left(t_{o}, u(t)\right)$. Also, since the quadratic cost function Eq. $(D 1-4)$ is a convex function, the $J_{\min }\left(x_{0}, u(t)\right)$ exists and is unique.

## Optimal Control Law

The optimal control problem can be solved by the aid of calculus of variations, or Pontryagin's maximum principle and/or Bellman's dynamic programming, Since there is no active constraint involved except the dynamical equation, the simplest straightforward way may be to use the calculus of variations. The result of this calculation is that matrix $K$ must
satisfy the "Algebraic Matrix Riccati Equation".

$$
\begin{aligned}
& \mathrm{KA}+\mathrm{A}^{\mathrm{T}} \mathrm{~K}-\mathrm{KDK}+\mathrm{H}^{\mathrm{T}} \mathrm{QH}=0 \\
& \text { (note } \mathrm{H}^{\mathrm{T}} \mathrm{QH} \mathrm{\geq 0)}
\end{aligned}
$$

where D is a $\mathrm{n} \times \mathrm{n}$ constant, symmetric, positive semidefinite matrix defined by

$$
\begin{equation*}
D \equiv B R^{-1} B^{T}(\geq 0) \tag{iD1-8}
\end{equation*}
$$

Let the matrix $K$ be a proper solution of Eq. (D1-7). Then the optimal control law is expressed in the linear feedback form. Using Eq. (DI-15) and (D1-17),

$$
\begin{align*}
u^{*}(t) & =-R^{-1} B^{T} K x^{*}(t) \\
& =-G x^{*}(t) \tag{D1-9}
\end{align*}
$$

where $G$ is the feedback gain matrix ( $\mathrm{m} \times \mathrm{n}$ ) defined by

$$
\begin{equation*}
G \equiv R^{-1} B_{K} T_{K} \tag{D1-10}
\end{equation*}
$$

By using the optimal feedback control law to the plant dynamics Eq. (D1-1), the optimal closed loop system dynamics become

$$
\begin{equation*}
\dot{x}(t) \equiv\left(A-B R^{-1} B^{T} K\right) x(t)=F x(t) \tag{D1-11}
\end{equation*}
$$

where the asterisks on $x(t)$ have been dropped off, and the $n x n$ matrix $F$ is the closed loop system matrix defined by

$$
\mathrm{F} \equiv \mathrm{~A}-\mathrm{BR}^{-1} \mathrm{~B}^{\mathrm{T}} \mathrm{~K} \equiv \mathrm{~A}-\mathrm{DK} \equiv \mathrm{~A}-\mathrm{BG}
$$

The eigenvalues of matrix $F$ indicate the poles of the optimal closed loop system. The stability of the system depends on the signs of the real parts of the eigenvalues. There exist many real symmetric solution matrices ( K 's) of the algebralc matrix Riccati equation (Eq. D1-7) for gıven positive definite matrices $Q$ and $R$. It has been shown by R.E. Kalman in Ref. (AI) that if [A, B] is a completely controllable pair, there exists a unique solution matrix K of Eq. ( $(\mathrm{ll} 19$ ), which is a $\mathrm{n} \times \mathrm{n}$ positive definite symmetric matrix and makes the real parts of all the eigenvalues of $F$ be negative (it means a stable system); namely,
$K>0$, unique and symmetric for $Q>0$ and $R>0$
and
$R_{e}\left\{\lambda_{i}[\cdot]\right\}$ means the real part of the $i-$ th eigenvalue of a matrix. Therefore, the closed loop system expressed by Eq. (D1-11) is asymptotically stable. Thus, the optimal control law $u^{*}(t)$ in Eq. (Dl-9) exists and is unique for given weighting matrices $Q$ and $R$.

It is important to note that the matrix Riccati equation (Eq. (D1-7) is Independent of the system state and time, and so is the optimal feedback gain matrıx $G$ in Eq. (D1-10).
observed vector) $y(t)$, despite the attempt to use the output $y(t)$ as the feedback signal. Hence, the optimal controller requires all the information about the system state $x(t)$, which has to be computable out of the observed vector $y(t)$. If the output vector $y(t)$ could not generate the (unique) state vector $x(t)$ or if the state vector $x(t)$ could not be directly observed (or measured), then the obtained optimal system would not be realizable.

A "state $x_{0}$ is observable at $t_{0}$ " if, given any control $u(t)$, there is a time $t_{1}>t_{0}$ such that knowledge of $u(t)$ and the output $y(t)$ over $\left[t_{0}, t_{1}\right]$ is sufficient to determine $x_{0}$. If the argument is true for every state $x_{0}$ at time $t_{0}$, then the system is said to be "observable at $t_{0}{ }^{"}$. If every $x_{0}$ is observable at every time $t_{o}$ in the interval of definition of the system, then the system is said to be "completely observable". Since the system under consideration here is stationary (constant coefficient), the system (Eqs. (DI-1), (DI-2), (D1-3)) is assumed to be completely observable.

The condition of complete observability for time-invariant linear system is that the following n-rows by np-columns composite matrix has the maximum rank ( $=\mathrm{n}$ )

$$
\begin{equation*}
\operatorname{Rank}\left[H^{T}: A^{T} H^{T}:\left(A^{T}\right)^{2} H^{T}: \ldots \ldots \cdot\left(A^{T}\right)^{n-1} H^{T}\right]=n \tag{D1-12}
\end{equation*}
$$

A matrix pair of $A$ and $H$ satısfying Eq. (D1-30) is said "Completely Observable pair $[A, H]^{\prime \prime}$.

## Minimum of Quadratic Cost Function

By using the unique optimal feedback control law (Eq. (D1-9)), the minimum of quadratic cost function $J_{\min }\left(x_{0}\right)$ can be uniquely evaluated in terms of $K$ matrix and the initial state $x_{0}$; and equals:

$$
\begin{equation*}
J_{\min }\left(x_{0} ; Q, R\right)=x_{0}^{T} K x_{0} \tag{1}
\end{equation*}
$$

Block Diagram Optimal Regulator
The block diagram of the optimal system described above is shown as Illustration D1-1.


Illustration D1-1. Block Diagram of Linear Optimal Control System

Observability
Eq. (pl-21) and the block diagram shows that the optimal control $u^{*} .(t)$ is a function of the state vector $x(t)$ rather than a function of the state vector $x(t)$ rather than a function of the output vector cor the

Note that, combining Eq. (D1-6) with Eq. (D1-12), the system is assumed to be both completely controllable and completely observable.

The observability condition Eq. (D1-12) guarantees that one can compute the unique state $x(t)$ from knowledge of the output $y(t)$ and the control $u(t)$ over $\left[t_{o}, t\right]$. When the output matrix $H$ is the $n x$ identity matrix ( $\left(I_{\mathrm{n}}\right.$ ), the problem is called "State-Regulator Problem", whereas the problem with a nonidentaty $H$ matrix is called "Output-Regulator Problem' ${ }^{\prime \prime}$.

Summary of the Deterministic Optimal Regulator
The deterministic optimal regulator for a completely controllable and observable system can be summarized as follows

System dynamics: $x(t)=A x(t)+B u(t) ; x\left(t_{0}\right)=x_{0}$ $y(t)=H x(t)$
where $x(t): n$ dimensional state vector $u(t): m$ dimensional control vector ( $m>n$ )
$y(t): p$ dimensional output vector ( $p>_{n}$ )
$A, B$ and $H: n x n, ~ n x m$ and $p x n$ constant matrices
Cost function: $\quad J_{\min }\left(x_{0}, Q, R\right)=\underset{u(t)}{\operatorname{Min}} \quad \underset{t_{0}^{f}}{f_{0}^{\infty}\left[y^{T}(t) Q y(t)+u^{T}(t) R u(t)\right] d t}$

$$
\begin{equation*}
=x_{0}^{T} K x_{0} \tag{4and11}
\end{equation*}
$$

where $Q$ and $R$ : positive definite symmetric constant matrices

Variables and equations:

Number of Variables
$x(t): n$
$u(t): m$
$y(t): p$
$J(\cdot): 1$
$Q: p^{2}$
(11) or (4) : m
(5) : $n^{2}$
$\mathrm{R}: \mathrm{m}^{2}$
$K: n^{2}$

Total: $\left(n+m+p+1+p^{2}+m^{2}+n^{2}\right)$ unknowns. Total: $\left(n+p+1+m+n^{2}\right)$ eqs. Conclusion: 1. If $Q, R$ are given, then number of variables $=$ number of equations.
2. If $Q$ and $R$ are not given, then the number of equations is $n^{2}+m^{2}$ less than the number of unknowns.

## APPENDIX D2

OPTIMAL LINEAR CONTROLLER DESIGN APPLIED TO A SECOND ORDER SYSTEM
The system equation is represented by the following differential equation

$$
\begin{equation*}
\ddot{x}(t)=a \dot{x}(t)+b u(t) \tag{D2-1}
\end{equation*}
$$

where

$$
\begin{aligned}
x(t) & =\text { position } \\
\dot{x}(t) & =\text { rate of position } \\
u(t) & =\text { control input } \\
a & =\text { rate damping } \\
b & =\text { control derivatives }
\end{aligned}
$$

In the state variable form

$$
\begin{equation*}
\dot{x}(t)=A x(t)+B u(t) \tag{D2-2}
\end{equation*}
$$

where

$$
\begin{align*}
& x(t)=\left[\begin{array}{l}
x_{1}(t)_{1} \\
x_{2}(t)_{2}
\end{array}\right]=\left[\begin{array}{l}
x(t) \\
\dot{x}(t)
\end{array}\right]  \tag{D2-3}\\
& A=\left[\begin{array}{ll}
0 & 1 \\
0 & a
\end{array}\right]  \tag{D2-4}\\
& B=\left[\begin{array}{l}
0 \\
b
\end{array}\right]  \tag{D2-5}\\
& u(t)=\delta \tag{D2-6}
\end{align*}
$$

## The Optimal Linear Regulator Approach

The quadratic performance index be minimized is:

$$
\begin{equation*}
J=\frac{1}{2} \int_{0}^{\infty}\left(x(t)^{T} Q x(t)+r u(t)^{2}\right) d t \tag{D2-7}
\end{equation*}
$$

where
$Q: 2$ by 2 non-negative, symmetrix weighting matrix for
$q_{11}, q_{12}, q_{12}, q_{21}>0$ states
$r:$ a scalar weighting factor for control

The optimal control which minimize (D2-7) is

$$
\begin{align*}
u & =-\operatorname{Gx}(t) \\
& =-\frac{1}{r} B^{T} K x(t) \tag{D2-8}
\end{align*}
$$

where $G$ is a feedback gain matrix and $K$ is a solution matrix of the following Riccati matrix equation.

$$
\begin{equation*}
K A+A^{T} k-\frac{1}{r} K B B^{T}+Q=0 \tag{D2-9}
\end{equation*}
$$

Expanding the latter equation gives:

$$
\frac{b^{2}}{r} k_{12} k_{21}-q_{11}=0
$$

$\frac{b^{2}}{r} k_{12} k_{22}-\left(a_{12}^{k}+k_{11}\right)-q_{12}=0$
$\frac{b^{2}}{r} k_{12} k_{22}-\left(a k_{21}+k_{11}\right)-q_{21}=0$
$\frac{\mathrm{b}}{\mathrm{r}} \mathrm{k}_{22} \frac{2}{}-2 \mathrm{ak}_{22}-\left(\mathrm{k}_{12}+\mathrm{k}_{21}\right)-\mathrm{q}_{22}=0$

From the symmetric property of weighting matrix $Q$ follows $q_{12}=q_{21}$ : The solution of Eq. $22-10$ becomes

$$
\begin{align*}
k_{11} & =\sqrt{q_{11} r\left\{\left(\frac{a}{b}\right)^{2}+\frac{2}{b} \sqrt{\frac{q_{11}}{r}}+\frac{q_{22}}{r}\right\}} \\
k_{12}=k_{21} & =\sqrt{\frac{q_{11} r}{b^{2}}}  \tag{D2-1I}\\
k_{22} & =\frac{r}{b^{2}}\left(a+\sqrt{\left.a^{2}+2 b \sqrt{\frac{q_{11}}{r}}+\frac{b^{2}}{r} q_{22}\right)}\right.
\end{align*}
$$

Substitution into (D2-8) gives the optimal control

$$
\begin{equation*}
u=-\frac{b}{r}\left[G_{x} x_{1}+G_{\dot{x}} x_{2}\right] \tag{D2-12}
\end{equation*}
$$

where $G_{x}$ is a position feedback gain and $G_{X}$ is its rate feedback gain given as follows:

$$
\begin{equation*}
G_{x}=-\sqrt{\frac{q_{11}}{r}} \tag{D2-13}
\end{equation*}
$$

$$
\begin{equation*}
G_{\dot{x}}=-\frac{a}{b}+\sqrt{\left(\frac{a}{b}\right)^{2}+\frac{2}{b} \sqrt{\frac{q_{11}}{r}}+\frac{q_{22}}{r}} \tag{D2-14}
\end{equation*}
$$

Note that the weighting factors $q_{12}$ or $q_{21}$ are irrelevant to the optimal feedback gains.

From ( $\mathrm{D} 2-2$ ) , ( $\mathrm{D} 2-12$ ), ( $\mathrm{D} 2-13$ ) and ( $\mathrm{D} 2-14$ ), the equation for the closed loop system becomes:

$$
\begin{equation*}
\dot{x}(t)+\sqrt{a^{2}+2 b \sqrt{\frac{q_{11}}{r}}+b^{2} \frac{q_{22}}{r}} \dot{x}+b \sqrt{\frac{q_{11}}{r}} x(t)=0 \tag{D2-15}
\end{equation*}
$$

When this closed loop system is described by

$$
\begin{equation*}
\ddot{x}(t)+2 \zeta \omega_{n} \dot{x}(t)+\omega_{n}^{2} x(t)=0 \tag{D2-16}
\end{equation*}
$$

then

$$
\begin{aligned}
& \omega_{\mathrm{n}}=\sqrt{\mathrm{b} \sqrt{\frac{q_{11}}{r}}} \\
& \zeta=\frac{1}{2} \sqrt{2+\frac{a^{2}}{b} \sqrt{\frac{r}{q_{11}}}+b q_{22} \sqrt{\frac{1}{\mathrm{rq}_{11}}}} \quad\left(q_{11} \neq 0\right) \quad \text { (D2-17) }
\end{aligned}
$$

Closed Loop System Properties as a Function of Control Weighting
When the control weighting $R$ approaches infinity, ( $R \rightarrow \infty$ )

$$
\begin{array}{rlrl}
G_{x} & =0 & \\
G_{\dot{x}} & =0 & & (a<0) \\
& =-\frac{2 a}{b} & & (a>0) \tag{D2-19}
\end{array}
$$

The feedback gains do not all go to zero; therefore the closed loop system
approaches the stable original system or the adjoint system in the case of an unstable original system because:

$$
\begin{equation*}
\ddot{x}(t)+|a| \dot{x}(t)=0 \tag{D2-20}
\end{equation*}
$$

always guarantees a stable system.
When the weighting for control approaches zero, ( $r=0$ ), which means there is no limitation on the control, the closed loop system approaches (12-21) which is called model control equation in (D2-1).

$$
\begin{equation*}
\sqrt{\frac{q_{22}}{q_{11}}} \dot{x}(t)+x(t)=0 \tag{D2-21}
\end{equation*}
$$

when the weighting factor $q_{22}=0$, the undamped natural frequency and damping become, from (D2-7):

$$
\begin{align*}
\omega_{n} & =\infty  \tag{D2-22}\\
\zeta & =.707
\end{align*}
$$

which are the second order Butterworth poles.

## The Minimum Value of the Performance Index

The minimum value of the performance index, in general, is obtained by substitution of the optimal control ( $\mathrm{D} 2-8$ ) into ( $\mathrm{P} 2-7$ ). This yields:

$$
\begin{equation*}
J_{\min }=\frac{1}{2} x_{0}^{T} K x_{0} \tag{D2-23}
\end{equation*}
$$

This is obtained by substituting D2-11 into the latter equation:

$$
\begin{aligned}
& J_{\min }=\frac{1}{2} x_{10}^{2}\left[\sqrt{\frac{q_{11} r}{b^{2}}\left(a^{2}+2 b \sqrt{\frac{q_{11}}{r}}+\frac{b^{2} q_{22}}{r}\right)}\right. \\
&+2 \sqrt{\frac{q_{11} r}{b^{2}}} \frac{x_{20}}{x_{10}} \\
&\left.+\frac{r}{b^{2}}\left(a+\sqrt{a^{2}+2 b \sqrt{\frac{q_{11}}{r}}+\frac{b^{2}}{r} q_{22}}\right)\left(\frac{x_{20}}{x_{10}}\right)^{2}\right] \\
&\left(x_{10} \neq 0\right)
\end{aligned}
$$

The formula, in general, is given as a function of initial states and solution of the Riccati equation. Minimizing thrs value is done by decreasing $r$ and also by decreasing $q_{11}$ and $q_{22}$. Therefore, without any constraints,minimizing $J$ itself is meaningless, Hence, constraints are necessary, which leads to the discussion of the OWEM method. (Appendix E).

## REFERENCE

D2-1 .Schultz, D.G. and Mersa, J.L., "State Functions and Linear Control Systems", McGraw Hill Inc., New York, 1967.

This appendix is a summary of the OWEM design method as presented in Refs. (E1) and (E2). The results of the method are given without proofs; for the detalls of mathematical derivations and proofs, see Ref. ( E1).

The goal of the linear optimal control theory is to find a linear control law such that the following quadratic cost function is minimized

$$
J(u(t) ; Q, R)=\underset{T \rightarrow \infty}{\operatorname{Lim}} \frac{1}{2 T} \int_{-T}^{+T}\left[y^{T}(t) Q y(t)+u^{T}(t) R u(t)\right] d t
$$

subject to

$$
\begin{aligned}
\dot{x}(t) & =A x(t)+B u(t)+C n(t) \\
y(t) & =H x(t) \\
\mathcal{E}\{n(t)\} & =0, \delta\left\{n(t) n^{T}(\tau)\right\}=N \delta(t-\tau)
\end{aligned}
$$

where $x(t): n$ dimensional state vector $u(t): m$ dimensional control vector
$y(t): p$ dimensional output vector
$n(t)$ : \& dimensional stationary, Gaussian, white nolse vector

Q : pxp output weighting matrix, weightings for the output errors, (positıve definite, symmetric, constant matrix)
$\mathrm{N} \quad$ : nxn noise intensity matrix
$\delta(\cdot):$ Dirac's delta function

$$
\begin{array}{cl}
R & : \text { mxm control weighting matrix (positive definite, } \\
& \text { symmetric, constant matrix) } \\
(\cdot)^{\mathrm{T}} & : \text { transpose of }(\cdot) \\
\mathcal{E}\{\cdot\} & : \text { expectation or average of }\{\cdot\}
\end{array}
$$

The optimal control law $\mathrm{u}^{*}(t)$ is obtained in the state feedback form (a linear function of $x(t)$ ) as follows

$$
\begin{equation*}
u^{*}(t)=-R^{-1} B^{T} K x(t) \equiv-G x(t) \tag{E1-2}
\end{equation*}
$$

where $(\cdot)^{-1}$ represents the inverse of ( $\cdot$ ), and $n \times n$ matrix $K$ is the unique posituve definlte solution of the following so-called "Algebraic Matrix Riccati (AMR)" equation:

$$
\begin{equation*}
K A+A^{T} K-K B R^{-1} B^{T} K+H^{T} Q H=0 \tag{E1-3}
\end{equation*}
$$

Thus, if the matrices $A, B, H, Q$, and $R$ are given, the controller can be designed by solving Eq. (E1-3) for $K$ and using it in Eq. $(E I=3)$.

Note that the optimal controller can be designed andependently of the noise intensity (covariance) and the matrix $C$ as long as the noise $n(t)$ is stationary, gaussian, white and of zero-mean.

Denote the corresponding state and output vectors to the optimal control $u^{*}(t)$ as $x^{*}(t)$ and $y^{*}(t)$ respectively. Namely,

$$
\begin{align*}
& \dot{x}^{*}(t)=A x^{*}(t)+B u^{*}(t)+\operatorname{Cn}(t)  \tag{E1-4}\\
& y^{*}(t)=H x^{*}(t) \tag{E1-5}
\end{align*}
$$

The equation should minimize $J(u(t) ; Q, R)$.

Assuming the stationarity of the system and the ergodic property, the minimum cost $J\left(u^{*}(t), Q, R\right)$ is obtained as follows (Ref. EI)

$$
\begin{align*}
J_{\min }(Q, R) & \equiv J\left(u^{*}(t) ; Q, R\right)=\mathcal{E}\left[y^{*}(t) Q y^{*}(t)+u^{*}(t) R u^{*}(t)\right) \\
& =\operatorname{Tr}\left[Q Y^{+}\right]+\operatorname{Tr}\left[R U^{*}\right]  \tag{E1-6}\\
& =\operatorname{Tr}\left[\mathrm{CNC}^{T} K\right] \tag{E1-7}
\end{align*}
$$

where $\operatorname{Tr}[\cdot]=$ Trace of $[\cdot]$
$=$ the sum of the diagonals of [.]
$Y^{*}(t)=\mathcal{E}\left\{y^{*}(t) y^{*}(t)\right\}$ : output covariance matrix
$U^{*}(t)=\mathcal{E}\left\{u^{*}(t) u^{*}{ }^{T}(t)\right\}:$ control covariance matrix
The matrices $\mathrm{A}, \mathrm{B}, \mathrm{C}$ and H are inherent to the given controlled system and normally available a priori. In order to design the controller, the immediate question to be asked is how to choose $Q$ and $R$ matrices. They affect the solution matrix $K$ in $A M R$ equation and thereby the feedback gain matrix $G\left(=R^{-1} B^{T} K\right)$. Hence, the closed loop characterıstics and the minimum quadratic cost $J_{m \perp n}(Q, R)$ have considerable effects of $Q$ and $R$.

First will be investigated how the minimum quadratic cost $J_{\min }(N ; Q, R)$ behaves with respect to $Q$ and $R$ :

Suppose there is an ordered sequence of trial weighting matrices Q.'s.

$$
\left[Q_{i}\right]=Q_{1}, Q_{2}, Q_{3}, \cdots Q_{i}, Q_{i+1} \cdots
$$

If $Q_{i+1}-Q_{i} \geq 0$ or positive semidefinite, then $\left[Q_{i}\right]$ is called a monotonically increasing definite sequence (MIDS). For an example, consider one $Q_{0}>0$ and a scalar sequence $\left[k_{i}\right] \equiv 1,2,3, \cdots, i, i+1, \cdots \cdots$. Then

$$
\begin{aligned}
Q_{i} & =k_{i} Q_{0} \\
{\left[Q_{i}\right] } & =Q_{1}, Q_{2}, Q_{3}, \cdots Q_{i}, Q_{i+1} \cdots \cdots \\
& =Q_{0}, 2 Q_{0}, 3 Q_{0}, \cdots i Q_{0},(i+1) Q_{0} \cdots \cdots
\end{aligned}
$$

is MIDS. Because

$$
Q_{1+1}-Q_{i}=(i+1) Q_{0}-i Q_{0}=Q_{0} \geq 0 \text { for all } i=1,2,3, \cdots
$$

Likewise, $\left[R_{i}\right]$ and $\left[K_{i}\right]$ are defined.
Using these special sequences, the parametric behavior of the minimum quadratic cost $J_{\text {min }}(N ; Q, R)$ is summarized below (Ref. [EI]).

A solution sequence $\left\{K_{i}\right\}$ of $A M R$ equation corresponds to the above defined sequences $\left[Q_{i}\right]$ and $\left[R_{i}\right]$.

Corresponding to the above defined sequence $\left[Q_{i}\right]$, and $\left[R_{1}\right]$ respectively, one obtains solution sequences $\left\{\mathrm{K}_{i}\right\}$ 's of $A M R$ equation. The solution sequences $\left\{\mathrm{K}_{\mathrm{i}}\right\}^{\prime}$ 's can be shown to be also monotonically increasing definite sequences $\left[K_{i}\right]$ 's. This is shown in Illustration E-1.



Illustration E-1. Behavior of $\left[K_{i}\right]$ with respect to $\left[\mathrm{O}_{\mathrm{i}}\right]$ and $\left[\mathrm{R}_{\mathrm{i}}\right]$

The $J_{\min }(N ; Q, R)$ with respect to a monotonically increasing definnte sequence $\left[K_{i}\right]$ also increases monotonically.


Illustration E-2. Behavior of $J_{\min }\left[K_{i}\right](N ; Q, R)$ with respect to $\left[K_{i}\right]$
The arrow in the illustration indicates the direction that is desirable for a design.

Define $J_{\min }\left(K_{i}\right) \triangleq \operatorname{Tr}\left[\mathrm{CNC}^{T} K_{i}\right] \equiv\left\{\begin{array}{l}J_{\min }\left(N ; Q_{i}, R\right) \\ \mathrm{om}_{\min }\left(N ; Q, R_{i}\right)\end{array}\right.$
Then if $K_{i+1}-K_{i} \geq 0$, then $J_{\min }\left(K_{i+1}\right) \geq J_{\min }\left(K_{i}\right)$

The behavior of $J_{\min }(N ; Q, R)$ with respect to $\left[Q_{i}\right]$ and $\left[R_{i}\right]$ is shown in Illustration E-3. The sequences $\left[Q_{i}\right]$ and $\left[R_{i}\right]$ are monotonically increasing definnte sequences.



$$
\text { Illustration E-3. } J_{\min } \text { with Respect to }\left[Q_{i}\right] \text { and }\left[R_{i}\right]
$$

The arrows in the illustration indicate the direction that is desirable in a design.

The monotonic property of $J(N ; Q, R)$ indicates that $J_{\text {min }}$ will not remain finite unless the weightang matrices $Q$ and $R$ are limited within some fincte allowable sets during their selection. When the elemental matrices of the allowable sets for choosing the $Q$ and $R$ matrices can be ordered and a definite differential matrix among any two of them is obtained; then the smallest weighting matrix yields the smallest $J_{\min }(N ; Q, R)$. Hence, the
allowable sets for the weighting matrices $Q$ and $R$ should not contain any matrices which can be ordered in the sense of definite differential matrix.

To bound $J_{m I n}(N ; Q, R)$ from both below and above, consider Eq. (El-6)

$$
J_{\min }(N ; Q, R)=\operatorname{Tr}\left[\mathrm{QY}^{*}\right]+\operatorname{Tr}\left[\mathrm{RU}^{*}\right]
$$

The general form to be bounded is

$$
\begin{aligned}
\operatorname{Tr}\left[\tilde{\mathrm{A}}^{\mathrm{T}} \tilde{\mathrm{~B}}\right] \quad \text { where } \quad & \tilde{\mathrm{A}}^{\mathrm{T}} \text { corresponds to } \mathrm{Q} \text { or } \mathrm{R} \\
& \tilde{B} \text { corresponds to } \mathrm{Y}^{*} \text { or } \mathrm{U}^{*} .
\end{aligned}
$$

This trace indicates a sum of the products:

$$
\begin{equation*}
\operatorname{Tr}\left[\tilde{A}^{T} \tilde{B}\right]=\sum_{i} \sum_{J} \tilde{a}_{i j} \tilde{b}_{i j} \tag{E1-10}
\end{equation*}
$$

where $\tilde{A}=\left\{\tilde{a}_{i j}\right\}, \tilde{B}=\left\{\tilde{b}_{i j}\right\}$.
For' a sum of productes of finite values, the lower bound is given by the arithmetic geometric average, the upperbound by the Cauchy-Schwarz theorem. For an example,

$$
\begin{equation*}
2 \sqrt{a_{1} a_{2} b_{1} b_{2}} \leq a_{1} b_{1}+a_{2} b_{2} \leq \sqrt{a_{1}^{2}+a_{2}^{2}} \sqrt{b_{1}^{2}+b_{2}^{2}} \tag{E1-11}
\end{equation*}
$$

where $a, b,>0$ and $a_{2} b_{2}>0$ for the left hand side inequality.
In matrix form this becomes [Ref.E1-3]:
When $\tilde{A}$ and $\tilde{B}$ are non singular ( $n \times n$ ) matrices then

$$
\begin{equation*}
\mathrm{n}|\tilde{A}|^{1 / n} \cdot|\tilde{B}|^{1 / n} \leq \operatorname{Tr}[\tilde{A} B] \leq||\tilde{A}|| \cdot| | \tilde{B}| | \tag{E1-12}
\end{equation*}
$$

where
$|\cdot|$ : Determinant
$\|\cdot\|:$ Matrix norm, $\|\tilde{\mathrm{A}}\|=\sqrt{\operatorname{Tr}\left[\tilde{\mathrm{A}}^{\mathrm{T}} \tilde{\mathrm{A}}\right]}$

The equalities occur at
the left hand side when $\tilde{A}^{T}=k_{1} \tilde{B}^{-1} \quad k_{1}>0$
the right hand side when $\tilde{A}^{T}=k_{2} \tilde{B} \quad k_{2}>0$.

Hence $J_{\min }(N ; Q, R)$ is bounded by

$$
\begin{align*}
& \mathrm{p}|\mathrm{Q}|^{1 / \mathrm{p}}\left|\mathrm{Y}^{*}\right|^{1 / \mathrm{P}}+\mathrm{m}|\mathrm{R}|^{1 / \mathrm{m}}|\mathrm{U}|^{1 / \mathrm{m}} \leq \mathrm{J}_{\min }(\mathrm{N} ; \mathrm{Q}, \mathrm{R}) \leq \\
& \leq\left\|Q| | \cdot| | \mathrm{Y}^{*}| |+\right\| \mathrm{R}\|\cdot\| \mathrm{U}^{*}| | \tag{E1-13}
\end{align*}
$$

Eq. E-12 indicates that:

1. When $|Q|$ and $|R|$ are prescribed, the minimum of $J_{\min }(N ; Q, R)$ with respect to $Q$ and $R$ can be obtained with $Q=k_{3} Y^{*-1}$ and $R=k_{4} U^{*-1}$.
2. When $\|Q\|$ and $\|R\|$ are prescribed, the maximum of $J_{\min }(N ; Q, R)$ with respect to $Q$ and $R$ can be obtained with $Q=k_{5} Y^{*}$ and $R=k_{6} U^{*}$. where $k_{3}, k_{4}, k_{5}$ and $k_{6}$ are appropriate positive scalars for $Q$ and $r$ to satisfy the prescribed determinants and norms.

The minimum and maximum values of $J_{\min }(N ; Q, R)$ occur when:
$\operatorname{Min} J_{\min }(N ; Q, R)$ when $Q=k_{3} Y^{*-1}$ and $R=k_{4} U^{*-1}$
$\operatorname{Max} J_{\min }(N ; Q, R)$ when $Q=k_{5} Y *$ and $R=k_{6} U^{*}$

The design method with Optimal Weighting Matrices (OWEM) is basically a linear optimal quadratic control system design method with calculated weighting factors such that the effect of disturbances on the performance index is minimized.

The OWEM design method can be summarized as follows:
The system is assumed to be completely controllable and observable and is shown in Illustration (E1-1).


Illustration El-1. Block Diagram of an Optımal Linear Regulator

$$
\text { System dynamics: } \begin{align*}
& \dot{x}(t)=A x(t)+B u(t)+C n(t)  \tag{E1-I4}\\
& y(t)=H x(t)  \tag{E1-15}\\
& \text { where } x(t): n \text { dimensional state vector } \\
& u(t): m \text { dimensional control vector }(\mathrm{m} \leq \mathrm{n}) \\
& y(t): p \text { dimensional output vector }(p \leq n) \\
& n(t): \ell \text { dimensional no1se vector }(\ell \leq n)
\end{align*}
$$

$$
\mathcal{E}\{n(t)\}=0, \operatorname{Cov}\{n(\tau)\}=\operatorname{NO}(t-\tau)
$$

$A, B, C$ and $H: n x n, n x m, n \times l$ and $p x n$ constant given matrices.
Cost function: $\left.J_{\min }(N ; Q, R)=\underset{u(t)}{\operatorname{Min}} \underset{T \rightarrow \infty}{\{\operatorname{Lim}} \frac{1}{2 T} \int_{-T}^{T}\left[y^{T}(t) O y(t)+u^{T}(t) R u(t)\right] d t\right\}$

$$
\begin{equation*}
=\operatorname{Tr}\left[\mathrm{CNC}^{\mathrm{T}} \mathrm{~K}\right] \tag{E1-17}
\end{equation*}
$$

where $Q$ and $R$ : positive definite symmetric matrices.
Optimal control law: $u(t)=-R^{-1} B^{T} K x(t)$
where $K$ : nxn unique positive definite solution of (E1-20)

$$
\begin{align*}
& \mathrm{KA}+\mathrm{A}^{T} K-K B R^{-1} B^{T} K+H^{T} Q H=0  \tag{E1-20}\\
& \text { Covarıance matrices: } \mathrm{FX}+X F^{T}+\mathrm{CNC}^{T}=0 \quad \text { (Ref. E2) }  \tag{EI-2I}\\
& \text { where } X=\mathcal{E}\left\{x(t) x^{T}(t)\right\}, F=A-B R^{-1} B^{T} K \\
& Y=\mathcal{E}\left\{y(t) y^{T}(t)\right\}=H X H^{T}  \tag{E1-22}\\
& U=\mathcal{E}\left\{u(t) u^{T}(t)\right\}=R^{-1} B^{T} K X K B R^{-1} \tag{E1-23}
\end{align*}
$$

In the above equations, the number of equations is $p^{2}+m^{2}$ less than the number of unknowns if $Q$ and $R$ are not given. To augment the lacking number of equations, an auxiliary performance index is minimized with respect to $Q$ and $R$.

Auxiliary performance index:

$$
\begin{align*}
& \operatorname{Min} J_{\min }(N ; Q, R) \equiv \underset{Q, R}{\operatorname{Min}} \operatorname{Tr}\left[C N C^{T} K\right]  \tag{E1-24}\\
& \text { subject to } K A+A^{T} K-K B R^{-1} B^{T} K+H^{T} Q H=0 \tag{E1-25}
\end{align*}
$$

Constraints on $Q$ and $R: \quad|Q|=1,|Q|=|R|=\rho^{2}$
Optimal weighting matrices $Q$ and $R$ :

$$
\begin{align*}
& Q=\sqrt[F]{\left|H P H^{T}\right| \cdot\left(\mathrm{HPH}^{T}\right)^{-1} \propto Y^{-1}}  \tag{E1-27}\\
& R=\rho^{2 / m} \frac{B^{T} K P K B}{\sqrt[m]{\left|B^{T} K P K B\right|}} \tag{E1-28}
\end{align*} U^{-1}
$$

where $\mathrm{FP}+\mathrm{PF}^{\mathrm{T}}+\mathrm{CNC}^{\mathrm{T}}=0 \rightarrow \mathrm{P} \equiv \mathrm{X}$ in this case.

$$
\begin{equation*}
F=A-B R^{-1} B^{T}{ }_{K} \tag{E1-29}
\end{equation*}
$$

## Conclusion:

If $\rho^{2}$ is given, then $Q$ and $R$ can be determined provided the norse intensity matrix $N$ is known. If the noise intensity matrix (or covariance matrix) $N$ cannot be estimated, then the approach in this sumnary cannot be used.

It has been shown in Ref. (E1) that the introduction of a new performance index independent of N can remove this deficiency. This performance Index extremizes the total system damping. It has been shown in Ref. [E1] that if one assumes that the disturbance is white noise entering the system at the controls with a magnitude $N=R^{-1}$ and $C=B$ then the results of both approaches yield the same result for the optimal calculation of the weighting matrix $Q$, however the optimal weighting matrix $R$ is different. Using the same system dynamics as above, with $\operatorname{Cov}\{n(t) ; n(\tau)=N \delta(t-\tau)$ unknown, one can summarize this approach as follows:

Cost function: $J_{\min }(N ; Q, R)=\underset{u(t)}{\operatorname{Min}} \quad\left\{\begin{array}{l}\operatorname{Lim}_{T \rightarrow \infty} \\ \frac{1}{2 T} \\ \\ -T\end{array} \quad \underset{-T}{T} \quad\left[\gamma^{T}(t) Q y(t)+u^{T}(t) R u(t)\right] d t\right\}$
$=\operatorname{Tr}\left[\mathrm{CNC}^{\mathrm{T}} \mathrm{K}\right]$. ( N : unknown)
If $\mathrm{N}_{\mathrm{e}}=k \mathrm{C}^{+} \mathrm{BR}^{-1} \mathrm{~B}^{\mathrm{T}}\left(\mathrm{C}^{+}\right)^{\mathrm{T}}$, then
$J_{\text {min }}\left(N_{e} ; Q, R\right)=k \operatorname{Tr}\left[B R^{-1} B^{T} K\right]$
where $C^{+}$: $\ell x n$ pseudo-inverse matrix of $C, i . e .$, $\left(=\left(C^{T} C\right)^{-1} C^{T}\right)$

Optimal control law: $u(t)=-R^{-1} B^{T} K x(t)$
where $K$ : nxn unique positive definite solution of $(E 1-21)$

$$
\begin{equation*}
K A+A^{T} K-K B R^{-1} B^{T} K+H^{T} Q H=0 \tag{E1-35}
\end{equation*}
$$

This optimal control law (22) can be computed without knowing $N$ if Q and R are given.

Covariance matrices (unknown):

$$
\begin{align*}
& F X+X F^{T}+C N C^{T}=0  \tag{E1-35}\\
& \text { where } X=\left\{x(t) x^{T}(t)\right\}, F=A-B R^{-1} B^{T} K  \tag{E1-37}\\
& Y=\left\{y(t) y^{T}(t)\right\}, H X H^{T} \\
& U=\left\{u(t) u^{T}(t)\right\}=E^{-1} B^{T} K X K B R^{-1}
\end{align*}
$$

Note: These covariances are all unknown since N is unknown
Auxiliary performance index:

$$
\begin{align*}
& \text { Total System Damping = TSD: the negative sum of the closed loop } \\
& \text { roots } \\
& \hat{L}^{*}=\underset{Q}{\operatorname{Min}} \underset{R}{\operatorname{Max}}\{T S D\}=\underset{Q}{\operatorname{Min}} \underset{R}{\operatorname{Max}}\left\{\operatorname{Tr}\left[-\left(A-B R^{-1} B^{T} K\right)\right]\right\}  \tag{E1-39}\\
& \text { or equivalently } \\
& L^{*}=\underset{Q}{\operatorname{Min}} \operatorname{Max}_{R}\left\{\operatorname{Tr}\left[B R^{-1} B^{T} K\right]\right\} \quad(\cdot A=\text { constant matrix }) \tag{E1-40}
\end{align*}
$$

subject to $K A+A^{T} K-K B R^{-1} B^{T} K+H^{T} Q H=0$
Note: (1) TSD is equivalent to the Relative Rate of Change of Error Sets (RERACES, see Ref. [1], Chapter IV), and is generalized system response speed.
(2) The extremum (minamax) of TSD does not exist unless $Q$ and $R$ are appropriately constrained.

Constraints on $Q$ and $R:|Q|=1,|R|=\rho^{2}$
where $|(\cdot)|:$ the determinant of $(\cdot)$
Optimal weighting matrices:

$$
\begin{align*}
& Q=\sqrt[p]{\left|H P H^{T}\right|} \cdot\left(H P H^{T}\right)^{-1}  \tag{E1-42}\\
& R=\rho^{2 / m} \cdot \frac{B^{T} K B-B^{T} K P K B}{\sqrt[m]{\left|B^{T} K B-B^{T} K P K B\right|}}  \tag{E1-43}\\
& \text { where } F P+{P F^{T}+B R^{-1} B^{T}=0}_{F}=A-B R^{-1} B^{T} K \tag{E1-44}
\end{align*}
$$

## Conclusions:

(1) When $N$ is unknown, $Q$ and $R$ can be computed by the minimaximization of TSD with respect to $Q$ and $R$ under prescribed determinant constraints.
(2) The optimal $Q$ matrix slows down the entire system response or error convergence speed by minimizing TSD. However, for an equivalent white noise distrubance with a magnitude $N_{e}$ inversely proportional to the control weighting $R$, the manimazation of TSD is equivalent to the manimization of $J_{\min }\left(N_{e} ; Q, R\right)$.

Note: a faster system response can be obtaned only at the expense of the entire system accuracy, i.e., a larger TSD accompanies a larger $J_{\min }(\cdot)$.
(3) The optimal $R$ is chosen such that the system response speed (with a given Q selection) is maximized.
(4) The optimal $Q$ matrix equalizes the closed loop poles in fre-quency-wise, while the optımal $R$ matrix tries to separate them. The trade-off between these two effects is controlled by $\rho^{2}$.
(5) $\rho^{2}$ is the only scalar parameter to be determined by designers.
(6) The approach summarized here can be directly applied for the deterministic case in Summary 1 since $Q$ and $R$ can be determined independently of N and C matrices.

Properties of the optimal weighting matrix determined by the OWEM design method (under determinant constraint), which can be of large significance in engineering applications, are the "Normalization and the Group-Wise Equalization".

Normalization
The state variables can be dimensionally different, and the weighted state and control variables are compared in the performance index. Hence, for a meaningful comparison, the weighted state and control variables should have the same dimensions. The resulting weighting matrices from the OWEM
design are inversely proportional to the output covariance matrix $Y$ and the optimal control covariance.

Group-Wise Equalization
In the development of the OWEM design the followingexpression was obtained:

$$
\begin{aligned}
Q^{+} & =\sqrt[P]{|Y|} \quad Y^{-1} \\
& =k_{5} Y^{-1} \quad \text { where } k_{5}=\sqrt[P]{|Y|} \quad(>0)
\end{aligned}
$$

Hence the $Q$ is inversely proportional to the covariance matrix $Y$.
The effect of the group-wise equalization constant $k_{5}$ can be evaluated as follows:

$$
\begin{gathered}
\mathcal{E}_{x}\left\{y^{*} T_{Q^{*} y^{*}}\right\}=T_{r}\left[Q^{*} E_{x}\left\{y^{*} y^{*} T_{\}}\right]=\sum_{i=1}^{P}\left(\sum_{j=1}^{P} q_{i j}^{*} \overline{Y_{j}^{*} Y_{i}^{*}}\right)\right. \\
Q^{*} \equiv\left\{q_{i j}^{*}\right\}=k_{3} Y^{*-1}=k_{3} \frac{\left\{Y_{j j}^{*}\right\}}{\left|Y^{*}\right|} \\
Y_{i j}^{*} \quad:(i-j) \text { th Cofactor of } Y^{*} \equiv\left\{\overline{y_{i}^{*} Y_{j}^{*}}\right\} \\
\sum_{j-1}^{P} q_{i j}^{*} \overline{Y_{j}^{*} Y_{i}^{*}}=k_{3} \sum_{j=1}^{P} \frac{Y_{j i}^{*}}{\left|Y^{*}\right|} \overline{Y_{j}^{*} Y_{i}^{*}}=k_{3} ; i=1,2, \cdots, p
\end{gathered}
$$

$$
\left.\begin{array}{l}
i=1: q_{11}^{*} \overline{y_{1}^{*} y_{1}^{*}}+q_{12}^{*} \overline{y_{2}^{*} y_{1}^{*}}+\cdots \cdots+q_{1 p}^{*} \overline{y_{p}^{*} y_{1}^{*}}=k_{3} \\
i=2: q_{21}^{*} \overline{y_{1}^{*} y_{2}^{*}}+q_{22}^{*} \overline{y_{2}^{*} y_{1}^{*}}+\cdots \cdots+q_{2 p}^{*} \overline{y_{p}^{*} y_{2}^{*}}=k_{3} \\
i=p: q_{p 1}^{*} \overline{y_{1}^{*} y_{p}^{*}}+q_{p 2} \overline{y_{2}^{*} y_{p}^{*}}+\cdots \cdots+q_{p p}^{*} \overline{y_{p}^{*} y_{p}^{*}}=k_{3}
\end{array}\right\} \begin{aligned}
& \text { p-groups }
\end{aligned}
$$

Similarly
$\sum_{i=1}^{p} q_{i j}^{*}\left(\overline{y_{j}^{\star} y_{i}^{*}}\right)=k_{3} \sum_{j=1}^{p} \frac{Y_{j i}^{*}}{\left|Y^{*}\right|} \overline{y_{j}^{\star} y_{i}^{*}}=k_{3} \quad j=1,2, \cdots p$.

This evaluation shows a group-wise equalization of the terms of $\left\{y *^{T} Q y^{*}\right\}$.

In the initial approach all the constants $\mathrm{k}_{3}$ are chosen to be equal. This corresponds to an Error Scaling Matrix (ESM) $=1$. Making the ESM matrix not equal to the identity matrix corresponds to an exror scaling factor (ESF) of the diagonal elements of the weighting matrix $q_{1 j}$.

The elements of the ESM matrix are diagonal, hence, an easier interpretation is often possible. In general, the elements corresponding to the quantities to be controlled determine the error weighting, while the elements corresponding to the derivatives of the quantities to be controlled influence the damping of that mode. In practical application, one should first use a ESM matrix with only elements that affect the quantity to be controlled. If mode damping increase is desirable, then elements of the ESM matrix that affect the mode damping can be introduced. This method, using a step by step approach, is preferable as the effects of the ESM matrix and $Q$ matrix are not yet fully investigated.

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A second order system equation is represented by the following differential equation

$$
\begin{equation*}
\ddot{x}(t)=a \dot{x}(t)+b u(t) \tag{E2-1}
\end{equation*}
$$

where

$$
\begin{aligned}
x(t) & =\text { position } \\
\dot{x}(t) & =\text { rate of position } \\
u(t) & =\text { control input } \\
a & =\text { rate damping } \\
b & =\text { control derivatives }
\end{aligned}
$$

In the state variable form

$$
\begin{equation*}
\dot{x}(t)=A x(t)+B u(t) \tag{E2-2}
\end{equation*}
$$

where

$$
\begin{align*}
x(t) & =\left[\begin{array}{l}
x_{1}(t) \\
x_{2}(t)
\end{array}\right]=\left[\begin{array}{l}
x(t) \\
x(t)
\end{array}\right]  \tag{E2-3}\\
A & =\left[\begin{array}{ll}
0 & 1 \\
0 & a
\end{array}\right]  \tag{E2-4}\\
B & =\left[\begin{array}{l}
0 \\
b
\end{array}\right]
\end{align*}
$$

$u(t)=\delta$

An analytical approach using the example clarifies the method and yields a solution for the choices of the $Q$ matrix. The closed loop system obtained by conventional optimal design method assigning values for $q_{11}, q_{22}$ and $r$ yields:

$$
\begin{equation*}
\ddot{x}(t)+\sqrt{a^{2}+2 b \sqrt{\frac{q_{11}}{r}}+b^{2} \frac{q_{22}}{r}} \dot{x}(t)+b \sqrt{\frac{q_{11}}{r}} x(t)=0 \tag{E2-7}
\end{equation*}
$$

The total system damping is (Ref. EI-I).

$$
\begin{equation*}
T S D=\sqrt{a^{2}+2 b \sqrt{\frac{q_{11}}{r}}+b^{2} \frac{q_{22}}{r}} \tag{E2-8}
\end{equation*}
$$

The determinant constraint on weighting matrix $Q$ becomes:

$$
\begin{equation*}
|Q|=1=q_{11} q_{22} \tag{E2-9}
\end{equation*}
$$

This determinant constraint only specifies the lower limit of the minimized performance index value. Note that for decreasing $q_{11}, q_{22}$ and $r$, the value of the performance index decreases and the minimized value becomes meaningless. The determinant constrant is used to avoid this. --
Substituting Eq.(E2-3) into Eq. (E2-2) reduces the TSD to

$$
\begin{equation*}
T S D=\sqrt{a^{2}+2 b \sqrt{\frac{q}{r}}+b^{2} \frac{1}{r q_{1 I}}} \tag{E2-10}
\end{equation*}
$$

The TSD value with respect to $q_{11}$ is illustrated in IIlustration (2-1).


Illustration 2-1. TSD as a Function of the Weighting $q_{11}$
. -
The minimization of TSD with respect to $q_{11}$ elements leads to minimization of the performance index with determinant constraint. The minimization of TSD is obtained by differentiating ( $\mathrm{E} 2-10$ ) with respect to $\mathrm{q}_{1.1}$. And TSD, $q_{11}$ and $q_{22}$ are:

$$
\begin{align*}
& T S D=\sqrt{a^{2}+\frac{3 b \frac{4}{3}}{r \frac{2}{3}}} \\
& q_{11}=\frac{b \frac{2}{3}}{r \frac{1}{3}}  \tag{E2-12}\\
& q_{22}=\frac{r \frac{1}{3}}{b \frac{2}{3}} \tag{E2-13}
\end{align*}
$$

Note that when the weighting factor $Q$ is obtained, the OWEM method minimizes the performance index.

$$
\begin{equation*}
J_{\min }=\frac{1}{2} \int_{0}^{\infty}\left(\frac{b^{\frac{2}{3}}}{r \frac{1}{3}} x_{1}(t)+\frac{r^{\frac{1}{3}}}{\frac{2}{3}} x_{2}(t)+r u^{2}(t)\right) d t \tag{E2-14}
\end{equation*}
$$

Substituting the optımal weighting factors into ( $\mathrm{E} 2-1$ ) gives the optimal closed loop system

$$
\begin{equation*}
\ddot{x}(t)+\sqrt{a^{2}+\frac{3 b \frac{4}{3}}{r \frac{2}{3}}} \dot{x}+\frac{b \frac{4}{3}}{r \frac{2}{3}} x(t)=0 \tag{E2-15}
\end{equation*}
$$

And the undamped natural frequency $\omega_{n}$ and the damping ratio $\zeta$ are respectively

$$
\begin{align*}
& \zeta=\frac{1}{2} \sqrt{3+\frac{\mathrm{a}^{2}}{\mathrm{~b} \frac{4}{3}} \frac{2}{2}}  \tag{E2-16}\\
& \omega_{\mathrm{n}}=\frac{\mathrm{b} \frac{2}{3}}{\mathrm{r} \frac{1}{3}} \tag{다2-17}
\end{align*}
$$

The optimal control is given by substituting ( $\mathrm{E} 2-12$ ) and ( $\mathrm{E} 2-15$ ) into ( $\mathrm{D} 2-13$ ), ( $\mathrm{D} 2-14$ ), and from ( $\mathrm{D} 2-12$ ).

$$
\begin{equation*}
u(t)=-\frac{b \frac{1}{3}}{r \frac{2}{3}} x(t)-\left(-\frac{a}{b}+\sqrt{\left(\frac{a}{b}\right)^{2}+\frac{3}{\left(b r \frac{2}{3}\right)}}\right) \dot{x}(t) \tag{E2-18}
\end{equation*}
$$

The solution of the Ricatti equation is as follows:

$$
\begin{align*}
k_{11} & =\sqrt{3+a^{2} \frac{r \frac{2}{3}}{b \frac{4}{3}}} \\
k_{12}=k_{21} & =\frac{r \frac{2}{3}}{b \frac{2}{3}}  \tag{E2-19}\\
k_{22} & =\frac{r}{b^{2}}\left(a+\sqrt{a^{2}+\frac{3 b \frac{4}{3}}{r \frac{2}{3}}}\right.
\end{align*}
$$

## Closed Loop System Properties as a Function of Control Weighting

When the control weighting $r$ goes to infinity, the closed loop system reduces the orıginal basic one or its adjoint one

$$
\begin{equation*}
\ddot{x}(t)+|a| \dot{x}(t)=0 \tag{E2-20}
\end{equation*}
$$

When the control weighting r goes to zero, then

$$
\begin{align*}
\omega_{\mathrm{n}} & =\infty  \tag{E2-2I}\\
\zeta & =0.866 \tag{E2-22}
\end{align*}
$$

which is more damped than that of the Butterworth poles for the second order system.

Minimized Value of the Performance Index
The minimized value of the performance index is:

$$
\begin{align*}
\frac{1}{2} J_{\min }= & \frac{1}{2}\left[k_{11} x_{10}^{2}+2 k_{12} k_{21} x_{10} x_{10}+k_{20}^{2}\right] \\
= & \frac{1}{2} x_{10}^{2}\left[\sqrt{3}+\mathrm{a} \frac{\mathrm{r} \frac{2}{3}}{\mathrm{~b} \frac{2}{3}}+\frac{r}{b^{2}}\left(a+\sqrt{a^{2}+\frac{3 b}{\frac{4}{3}}} \frac{x_{20}^{3}}{x_{10}}{ }^{2}\right.\right. \\
& \left.+\frac{2 r \frac{2}{3}}{\mathrm{~b} \frac{2}{3}} \frac{x_{10}}{x_{10}}\right] \tag{E2-23}
\end{align*}
$$

when $r=0$,

$$
\begin{equation*}
\frac{1}{2} J_{\operatorname{mIn}}=\frac{\sqrt{3}}{2} x_{10}{ }^{2} \tag{E2-24}
\end{equation*}
$$

Note that its value for conventional selection is given by

$$
\frac{1}{2} \mathrm{~J}_{\mathrm{min}}=\frac{1}{2} \sqrt{\mathrm{q}_{11} \mathrm{q}_{22}} \times 10^{2}
$$

which becomes zero when $\mathrm{q}_{22}=0$ -

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## THE ROOT SQUARE LOCUS

The Root Square Locus Method

In the optimal control problem is minimized

$$
\begin{equation*}
J=\frac{1}{2} \int_{0}^{\infty}\left(y Q_{y}+u^{T} R_{u}\right) d t \tag{F1-1}
\end{equation*}
$$

subject to

$$
\begin{align*}
& x(t)=A x(t)+B u  \tag{F1-2}\\
& Y=H x, H=I \tag{F1-3}
\end{align*}
$$

where $x(t)$ : $n$ dimensional state vector
$\mathrm{u}(\mathrm{t})$ : in diemsnioanl control vector ( $\mathrm{m}<\mathrm{n}$ )
$y(t): p$ dimensional output vector $(p \leq n)$
$\mathrm{A}, \mathrm{B}$ and $\mathrm{H}: \mathrm{nxn}, \mathrm{nxm}$ and pxn constant matrices

The solution of this problem yields the optimal control

$$
\begin{equation*}
u^{*}=-R^{-1} B^{T} K x(t) \tag{F1-4}
\end{equation*}
$$

where $K$ is a solution of the Riccati matrix equation

$$
\begin{equation*}
K A+A^{T} K-K B R^{-1} B^{T} K+Q=0 \tag{F1-5}
\end{equation*}
$$

The optimal closed loop system is given by

$$
\begin{align*}
\dot{x}(t) & =A x(t)+B u^{*}(t) \\
& =A x(t)+B\left(-R^{-1} B^{T} K\right) x(t)  \tag{F1-6}\\
& =(A-G K) x(t) \tag{F1-7}
\end{align*}
$$

where

$$
\begin{equation*}
G=B R^{-1} B^{T} \tag{F1-8}
\end{equation*}
$$

The characteristic equation is:

$$
\begin{equation*}
|s I-A+B K|=0 \tag{F1-9}
\end{equation*}
$$

The roots of the characteristic equation can be obtained by solving (Eq. F1-9) with a prior1-known $K$. The relationship between $K, Q$ and $R$ is given by (Eq. FI-5), and is non-1inear.

The spectral form product of the characteristic Eq. F1-9

$$
\begin{equation*}
|s I-A+B K|\left|-S I-(A-B K)^{T}\right|=0 \tag{Fl-10}
\end{equation*}
$$

The characteristic equations of the optimal system and its adjoint

$$
\begin{align*}
& \dot{x}-A x+B R^{-1} B^{T} \lambda=0  \tag{F1-11}\\
& H^{T} Q H x+\dot{\lambda}+A^{T} \lambda=0
\end{align*}
$$

$$
\left|\begin{array}{cc}
\mathrm{IS}-\mathrm{A} & \mathrm{BR}^{-1} \mathrm{~B}^{T}  \tag{F1-1.2}\\
-\mathrm{H}^{T} \mathrm{QH} & -\mathrm{IS}-\mathrm{A}^{T}
\end{array}\right|=0
$$

and the following relationship is satisfied.

$$
|S I-A+B K|\left|-S I-(A-B K)^{T}\right|=\left|\begin{array}{ll}
I S-A & B R^{-1} B^{T} \\
-H^{T} Q H & -I s-A^{T}
\end{array}\right|
$$

The roots can be obtained by solving (Eq. F1-12) with respect to $Q$ and $R$ or graphically by making the "root square locus". Equation F1-13 yields expressions relating the feedback gains $K$ with $A, B, H$, and $Q$.

Adding the control equation to (Fl-11) one obtains:

$$
\begin{align*}
& \dot{x}-A x+B^{-1} B^{T} \lambda=0 \\
& H^{T} Q H+\dot{\lambda}+A^{T} \lambda=0  \tag{F1-14}\\
& R u+B^{T} \lambda=0
\end{align*}
$$

and substituting $u=-R^{-1} B^{T} \lambda$ into the top equation of (F1-14), we can obtain the characteristic equation,

$$
\left|\begin{array}{ccc}
-I s-A^{T} & -H^{T} Q H & 0  \tag{F1-15}\\
0 & I s-A & -B \\
B^{T} & 0 & R
\end{array}\right|=0
$$

In the transfer function form, this becomes:

$$
\begin{equation*}
\left|I+R^{-1}\left[\frac{Y}{u}(-s)\right]^{T} Q\left[\frac{r}{u}(s)\right]\right|=0 \tag{F1-16}
\end{equation*}
$$

where

$$
\begin{align*}
\frac{y}{u}(s) & =H(I s-A)^{-1 B} \\
& =\left[\begin{array}{llll}
\frac{y_{1}}{u_{1}}(s) & \frac{y_{1}}{u_{2}}(s) & \cdots & \frac{y_{1}}{u_{m}}(s) \\
\frac{y_{p}(s)}{u_{1}(s)} & \frac{y_{p}}{u_{2}}(s) & \ldots & \frac{y_{p}}{u_{m}}(s)
\end{array}\right] \tag{F1-17}
\end{align*}
$$

For single input system with $R$ equal to a scalar $\rho$, the characteristic equation becomes:

$$
I+\rho^{-2}\left[\frac{y}{u}(-s)\right]^{T} Q\left[\frac{y}{u}(s)\right]=0
$$

where

$$
\frac{y_{1}}{u}(s)=\left[\begin{array}{c}
\frac{y_{1}}{u}  \tag{F1-18}\\
\frac{y_{2}}{u} \\
\frac{y_{p}}{u}
\end{array}\right]
$$

The root square locus can be made by using similar rules as used for the root locus method, e.g.:
$N$ number of poles of $\frac{Y}{u}(s)$
$p$ number of zeroes of $\frac{y}{u}(s)$

Angle condition:

```
(N-p) = odd number 0 degree condition
(N-p) = even number 180
```

Asymptotes:

$$
\begin{aligned}
& \frac{2 n \pi}{2(N-p)} \quad N-p=\text { odd } \quad n=0, \pm 1, \pm 2, \quad 2(N-p)-1 \\
& \frac{2 n \pi}{2(N-p)} \quad N-p=\text { even } \quad n=0, \pm 1, \pm 2, \quad 2(N-p)
\end{aligned}
$$

Center of asymptotes: always at origin

$$
\begin{aligned}
& 2(N-p) \text { poles go to infinity along asymptotes, as } p^{2} \rightarrow 0 \\
& 2(p) \text { poles to zero, as } \mathrm{p}^{2} \rightarrow 0
\end{aligned}
$$

## APPENDIX F2

ROOT SQUARE LOCUS METHOD APPLIED TO A HELICOPTER (HOVER)

The following is a summary from [Ref. Fl ]. Consider a single input, dynamic system of the form:

$$
\begin{equation*}
\dot{y}(t)=A y(\tau) \geqslant B u(\tau) \tag{F2-1}
\end{equation*}
$$

where $y(c)=$ n-dimensional state vector $u(c)=m$-dimensional control vector $A, B=n x n$ and nxm constant matrices

For an optimal control Iaw, we seek to minimize the cost function.

$$
J(Q, R)=\int_{c_{0}}^{\infty}\left[\mathbb{R}^{T}(\tau) Q y(t) \div u^{T}(t) R u(t)\right] d t(F 2-2)
$$

where $Q, R=$ positive definite symmerric constant weighting macrices.

For this analysis, $\mathbb{Q}$ is chosen co be:

$$
Q=\left[\begin{array}{cccc}
q_{11} & 0 & 0 & 0  \tag{F2-3}\\
0 & q_{22} & 0 & 0 \\
0 & 0 & q_{33} & 0 \\
0 & 0 & 0 & q_{44}
\end{array}\right]
$$

and $\mathrm{R}=\mathrm{p}^{2}$, a scalar

The performance index (cost function) then becomes

$$
\begin{equation*}
J=\frac{1}{2} \int_{0}^{\infty}\left(\dot{y} T \mathrm{~T} \mathbf{Q}+u^{T} \rho^{2} u\right) d t \tag{F2-5}
\end{equation*}
$$

The cnaracteristic equarion of the single input syscem is shows to be (Appendix F1).

$$
\begin{equation*}
I+\rho^{-2}\left[\frac{y}{u}(-s) T\right] Q\left[\frac{y}{u}(s)\right]=0 \tag{F2-6}
\end{equation*}
$$

The characteristic equation then becomes:

$$
1+\frac{1}{\rho^{2}}\left[\frac{y_{1}}{u}(-s), \frac{y_{2}}{u}(-s), \frac{y_{3}}{u}(-s), \frac{y_{4}}{u}(-s)\right]\left[\begin{array}{cccc}
q_{11} & 0 & 0 & 0  \tag{F2-7}\\
0 & q_{22} & 0 & 0 \\
0 & 0 & q_{33} & 0 \\
0 & 0 & 0 & q_{4}
\end{array}\right]\left[\begin{array}{l}
\frac{y_{2}}{u}(s) \\
\frac{y_{2}}{u}(s) \\
\frac{y_{2}}{u}(s) \\
y_{y_{4}} \\
u
\end{array}\right]=0
$$

The change in roors of 'this equation, resulting from varying a given parameter may be described by a root-square locus. The root-square locus technique is described in (Appendix F1).

A simplified longitudinal model for a helicopter is:

$$
\begin{align*}
& \left(s-X_{u}\right) u+g \theta=X_{B_{1}}{ }^{B_{1}} s  \tag{F2-8}\\
& -M_{u} u+s\left(s-M_{q}\right) \theta=M_{B_{1}}{ }^{B} 1_{s} \tag{F2-9}
\end{align*}
$$

and may be expressed in state variable form as:

$$
\frac{d}{d t}\left[\begin{array}{l}
y_{1}  \tag{F2-10}\\
y_{2} \\
y_{3} \\
y_{4}
\end{array}\right]=\left[\begin{array}{cccc}
0 & 1 & 0 & 0 \\
0 & x_{u} & -g & 0 \\
0 & 0 & 0 & 1 \\
0 & m_{u} & 0 & M_{q}
\end{array}\right]\left[\begin{array}{l}
y_{1} \\
y_{2} \\
y_{3} \\
y_{4}
\end{array}\right] \quad+\left[\begin{array}{c}
0 \\
x_{B_{1}} \\
0 \\
M_{B_{1}}
\end{array}\right] \mathrm{u}
$$

where

$$
\left[\begin{array}{l}
y_{1}  \tag{F2-11}\\
y_{2} \\
y_{3} \\
y_{4}
\end{array}\right]=\left[\begin{array}{c}
x \\
\dot{x} \\
\theta \\
\dot{\theta}
\end{array}\right], \quad u=B_{1}
$$

From Eq. (F2-7) , the characteristic equation is

$$
\begin{align*}
1+\frac{1}{\rho^{2}}\left[\left(q_{11} \frac{y_{1}}{u}(-s) \frac{y_{1}}{u}(s)\right.\right. & +q_{22} \frac{y_{2}}{u}(-s) \frac{y_{2}}{u}(s) \div q_{33} \frac{y_{3}}{u}(-s) \frac{y_{3}}{u} \\
& \left.+q_{44} \frac{y_{4}}{u}(-s) \frac{y_{4}}{u}(s)\right]=0 \quad(F 2-1) \tag{F2-12}
\end{align*}
$$

where $\frac{y_{1}}{u}, \frac{y_{2}}{u}, \frac{y_{3}}{u}$, and $\frac{y_{4}}{u}$ are given by:

$$
\begin{align*}
& \frac{y_{1}}{u}=\frac{x}{u}=\frac{X_{B_{1 s}}\left[s^{2}-M_{q} s-\left(\frac{M_{B_{1 s}}}{X_{B_{1 s}}}\right) g\right]}{s\left[s^{3}+\left(X_{u}+M_{q}\right) s^{2}+X_{u} M_{q} s+g M_{u}\right]}  \tag{F2-13}\\
& \frac{y_{2}}{u}=\frac{\dot{x}}{u}=\frac{X_{B_{1 s}}\left[s^{2}-M_{q} s-\left(\frac{M_{B}}{X_{1 s}}\right) g\right.}{\left[s^{3}-\left(X_{u}+M_{q}\right) s^{2} \div X_{u} M_{q} s+g M_{u}\right]} \tag{F2-14}
\end{align*}
$$

$$
\begin{align*}
& \frac{y_{3}}{u}=\frac{\theta}{u}=\frac{M_{B_{1 s}}\left[s-x_{u}+\left(\frac{{ }^{X_{B}}{ }_{1 s}}{M_{B_{1 s}}}\right) g\right]}{\left[s^{3}-\left(X_{u}+M_{q}\right) s^{2}+X_{u}{ }^{n}{ }^{\prime} s+g M_{u}\right]}  \tag{F2-15}\\
& \frac{y_{u}}{u}=\frac{\dot{\theta}}{u}=\frac{{ }_{B}{ }_{B_{1 s}} s\left[s-X_{u}+\left(\frac{x_{B_{1 s}}}{M_{B_{1 s}}}\right) g\right]}{\left[s^{3}-\left(x_{u}+M_{q}\right) s^{2}+x_{u q} M_{q} s+g M_{u}\right]} \tag{F2-16}
\end{align*}
$$

In order to determine the effect of $\mathrm{q}_{\mathrm{ii}}$ on the roots of Equation F2-12, we apply the root-square locus technique. These cases will be examined. First, weighting in the performance index will be applied only to $\bar{y}_{1}$, position. Next, the performance index will weight position and velocity. Weighting will then be applied to position, velocity and attitude. Then all four state variables will be weighted.

Finally,is given a welghting for position and attitude.
Case I

$$
\begin{align*}
& \text { Characteristic equation: } 1+\frac{q_{11}}{\rho} \frac{y_{1}}{u}(-s) \frac{y_{1}}{u}(s)=0 \\
& J=\frac{1}{2} \int_{0}^{\infty}\left(q_{11} y_{1}^{2}+\rho{ }^{2} u^{2}\right) d t \tag{F2-17}
\end{align*}
$$

The root-square locus is shown in I1lustration F2-1.

## Case II

Characteristic equation: $1+\frac{q_{11}}{\rho^{2}} \frac{y}{u}(-s) \frac{y}{u}(s)+\frac{q_{22}}{\rho^{2}} \frac{y_{2}}{u}(-s) \frac{y_{2}}{u}(s)$
or:

$$
\begin{align*}
& \therefore 1+\frac{q_{12}}{\rho^{2}} \frac{y(s)}{u}=\frac{y(-s)\left(1-s^{2} \frac{q_{22}}{q_{11}}\right)=0}{} \begin{array}{l}
J=\frac{1}{2} \int_{0}^{\infty}\left(q_{11} y_{1}^{2}+q_{22} y_{2}^{2}+\rho^{2} u^{2}\right) d t
\end{array}, l \tag{F2-20}
\end{align*}
$$

The root-square locus is shown in Illustration F2-2.

## Case IIIa

Characteristic equation:

$$
1+\frac{q_{11}}{\rho^{2}} \frac{y_{1}}{\frac{u}{u}}(s) \frac{y_{1}}{u}(-s)+\frac{q_{33}}{\rho^{2}} \frac{y_{3}}{u}(s) \frac{y}{u^{3}}(-s)=0
$$

$$
1+\frac{q_{11}}{\rho^{2}} \frac{y_{1}}{u}(s) \frac{y_{1}}{u}(-s) .
$$

$$
\left[1+\frac{q_{33}}{q_{11}}\left(-s^{2}\right) \frac{M_{B_{1 s}}^{2}}{X_{B_{1 s}}^{2}} \frac{\left(s-X_{u}+\frac{X_{B_{1 s}}}{M_{B_{1 s}}} M_{u}\right)\left(-s-X_{u}+\frac{X_{B_{1 s}}}{M_{B_{1 s}}} M_{u}\right)}{\left(s^{2}-M_{q} s-\frac{M_{1 s}}{X_{B_{1 s}}} g\right)\left(s^{2}+M_{q} s-\frac{M_{1 s}}{X_{B_{1 s}}} g\right)}\right]=0
$$

$$
(F 2-23)
$$

$$
\begin{equation*}
J=\frac{1}{2} \int_{0}^{\infty}\left(q_{11} y_{1}^{2}+q_{33} y_{3}^{2}+\rho^{2} u^{2}\right) d t \tag{F2-24}
\end{equation*}
$$

The root square locus is shown in Illustration $\mathrm{F} 2-3$.

## Case LIb

Characteristic equation:

$$
1+\frac{q_{11}}{\rho^{2}} \frac{y_{1}}{u}(-s) \frac{y_{1}}{u}(s)+\frac{q_{22}}{\rho^{2}} \frac{y_{2}}{u}(-s) \frac{y_{2}}{u}(s)+\frac{q_{33}}{\rho^{2}} \frac{y_{3}}{u}(-s) \frac{y_{3}}{u}(s)=0
$$

or

$$
\begin{aligned}
& 1+\frac{q_{11}}{\rho^{2}} \frac{y_{1}}{u}(s) \cdot \frac{y_{1}}{u}(-s)\left(1+\frac{q_{22}}{q_{11}}\left(-s^{2}\right)\right)
\end{aligned}
$$

$$
\begin{align*}
& =0 \\
& \text { (F2-25) } \\
& \text { The root-square locus is shown in Illustrations F2-4. }
\end{align*}
$$

Case IV
Characteristic equation:

$$
\begin{align*}
& I+\frac{q_{11}}{\rho^{2}} \frac{y_{1}}{u}(-s) \frac{y_{1}}{u}(s)+\frac{q_{22}}{\rho^{2}} \frac{y_{2}}{u}(-s) \frac{y_{2}}{u}(s)+\frac{q_{33}}{\rho^{2}} \frac{y_{3}}{u}(-s) \frac{y_{3}}{u}(s) \\
& +\frac{q_{44}}{\rho^{2}} \frac{y_{4}}{u}(-s) \frac{y_{4}}{u}(s)=0 \tag{F2-26}
\end{align*} \quad(F 2-26) \quad l l
$$

or:

$$
\begin{aligned}
& 1+\frac{q_{11}}{\rho^{2}} \frac{y_{1}}{u}(s) \frac{y_{1}}{u}(-s)\left(1+\frac{q_{22}}{q_{11}}\left(-s^{2}\right)\right) . \\
& {\left[1+\frac{q_{33}}{q_{11}}\left(-s^{2}\right) \frac{M_{B_{1 s}}^{2}\left(s-X_{u}+\frac{X_{B_{1 s}}}{M_{B_{1 s}}} M_{u}\right)\left(-s-X_{u}+\frac{X_{B_{1 s}}}{M_{B_{1 s}}} M_{u}\right)\left(1+\frac{q_{44}}{q_{33}}(-s)\right.}{X_{1 s}^{2}\left(s^{2}-M_{q} s-\frac{M_{B_{1 s}}}{X_{B_{1 s}}} q\right)\left(s^{2}+M_{q} s-\frac{M_{B_{1 s}}}{X_{B_{1 s}}} g\right)\left(1+\frac{q_{22}}{q_{11}}\left(-s^{2}\right)\right.}\right]} \\
& =0 \\
& \text { (F2-27) }
\end{aligned}
$$

The root square locus is shown in Illustrations (F2-5).

## Conclusions

1. In Case I, called position error control, where weighting is applied only to position in the performance criterion, the short period mode is not highly damped and the best possible phugoid damping is .707 .
2. In Case II, called velocity error control, where weighting is applied to both position and velocity, the damping of the attitude mode is still not high, but the position mode can be more damped than in Case I.
3. In Case III, called acceleration error control, where weighting is applied to position, velocity and attitude, the damping of the attitude mode can be made greater than e1ther of the two preceding cases.
4. In Case IV, called accurate rate control, where weighting is applied to position, velocity, attitude, and attitude rate, the attitude mode is further damped.
5. For precision flight path control with minimum (position) error, the velocity exror control is minimum desirable.
6. Acceleration and acceleration rate error control do not have a large effect upon the position error, but yield a reduction in control effort.
7. In conclusion, for precision flight path control, only the short term dynamics for velocity, acceleration and accurate rate are important, hence one needs to consider only the "dynamic error control" for precision in flight path control.


Ilustration F2-1. Root-Square Locus - Weighting Applied to $q_{11}$, Position With varying $\frac{q}{} \frac{11}{\rho^{2}}$
the zeros axe determined by the zeros of $\frac{X_{1}}{u}$ or $\frac{X}{B_{1 s}}$ )


Illustration F2-2. Root Square Locus - Weighting Applied to $q_{11}$ (Position) and $a_{22}$ (Velocity) with varying $q_{22}$


Illustration F2-3. Root Square Locus Weighting Applied to $q_{11}$ (position) and $q_{33}$ (attitude) with Varying $q_{33}$


Illustration F2-4. Root Square Locus Weighting Applied to $q_{11}$ (Position), $q_{22}$ (Velociry) and $q_{33}$ (Attitude) with varying $\frac{q_{33}}{\rho^{2}}$


Illustration F2-5. Root Square Locus Weighting Applied to $q_{11}$ (Position), $q_{22}$ (Velocity), $\mathrm{q}_{33}$ (Atticude), and $\mathrm{q}_{44}$ (Atritude Rate)

## APPENDIX G1

REVIEW OF MODEL FOLLOWING BY LINEAR OPTIMAL CONTROL

## Introduction

Given the system:

$$
\begin{equation*}
\dot{x}(t)=A x(t)+B u(t) \tag{G1-1}
\end{equation*}
$$

where: $\quad x(t): n$ dimensional state vector, and output vector
$u(t): m$ dimensional control vector (mn)
A, $B: n \times n$, nxm and pxn constant matrices
The "ideal" model is given by:

$$
\begin{equation*}
y(t)=A_{m} y(t) \tag{G1-2}
\end{equation*}
$$

where:
$A_{m}$ is nxn constant matrix of the model

There are in principle two methods to provide a good match between the dynamic of the system and the dynamic of the "ideal model". Namely, 1. model in the perfomance index and 2 . model in the system.

Model in the Performance Index
This approach is shown in Illustration G1 and uses the minimization of the norm:

$$
\begin{equation*}
\left\|\dot{x}-A_{m} x(t)\right\| \tag{G1-3}
\end{equation*}
$$



Illustration G1-1. Mode1 Following (Model in the Performance Index)

Hence, the following performance index is defined.

$$
\begin{equation*}
2 V=\operatorname{Lim}_{T \rightarrow \infty} \int_{o}^{T}\left(\dot{x}(t)-A_{m} x(t)\right)^{T} Q\left(\dot{x}(t)-A_{m} x(t)\right)+u^{T}(t) \operatorname{Ru}(t) d t \tag{G1-4}
\end{equation*}
$$

Using the Hamiltonian and maximum principle of Pontryagin (Ref. [G1-2], [G1-9]), one obtains the optimal control law:

$$
\begin{equation*}
u(t)=-\left(\hat{R}^{-1} \hat{B}^{T} P+\hat{R}^{-1} \hat{B}^{T} Q\left(A-A_{m}\right) \quad x(t)\right. \tag{G1-5}
\end{equation*}
$$

where $P$ is the $n \times n$ steady state solution to the Riccati equation:

$$
\begin{equation*}
-P \hat{B R}^{-1} \hat{B}^{T} p+A^{T} P+\left(A-A_{m}\right)=0 \tag{G1-6}
\end{equation*}
$$

and

$$
\begin{align*}
& \hat{R}=B^{T} Q B+R \\
& \hat{A}=A-B R^{-1} B^{T} Q\left(A-A_{m}\right)  \tag{Gl-7}\\
& \hat{B}=B
\end{align*}
$$

When the dimensions of the model matrix are the same as that of system matrix and when the system and model are exactly matchable (Ref. G2 and G9), the optimal control is of the following form:

$$
\begin{equation*}
u^{*}(t)=-\left(A-A_{m}\right) x(t) \tag{G1-8}
\end{equation*}
$$

Substituting Eq. (G1-6) into (G1-1) the optimal closed loop becomes:

$$
\begin{equation*}
x(t)=A_{m} x(t) \tag{G1-9}
\end{equation*}
$$

which has the dynamic of the model. This exact matching is achieved (in general) as $|Q| /|R|$ becomes large. (Ref. [G1-2]).

When the model and system are exactly matchable, there is no need in using optimal control and the optimal control * $(t)$ can be derived from algebraic considerations (Ref. [G1-6]). The algebraic conditions for "perfect"
 and system be the same, it is required that:

$$
\begin{equation*}
x(t)=A_{m} x(t) \tag{G1-10}
\end{equation*}
$$

Substituting (G1-8) into G1-1),

$$
\begin{equation*}
A_{m} x(t)=A x(t)+B u(t) \tag{G1-11}
\end{equation*}
$$

hence

$$
\begin{equation*}
B u(t)=\left(A_{m}-A\right) x(t) \tag{G1-12}
\end{equation*}
$$

Multiplying both sides of (G1-10) by $B^{+}$(where $B^{+}$is the pseudoinverse of $B$ ) one obtains

$$
\begin{equation*}
u(t)=B^{+}\left[A_{m}-A\right] x(t) \tag{G1-13}
\end{equation*}
$$

It has been shown in Ref. [G1-6] that necessary and sufficient conditions for perfect matches between the dynamics of the plant and that of the model are:

$$
\begin{equation*}
\left[B B^{+}-I\right]\left[A_{m}-A\right]=0 \tag{G1-14}
\end{equation*}
$$

This equation indicates that for $u(t)$ (given by G1-13) to exist, there must be a great structural similarity between the system matrix $A$ and the model matrix L. Moreover, B must have sufficient rank. However, it is clear that for this limited class of problems, this approach involves much less computation than the quadratic cost integral approach.

Model in the System
This approach is shown in Illustration $G 2$ and is based on the minimization of the square error between the output of the given plant and the output of the
desired plant model; thus forcing the output of the given system to follow the model.


Illustration G1-2. Model Following (Model in the System)

Given the system:

$$
\begin{equation*}
\dot{x}_{s}(t)=A_{s} x_{s}(t)+B_{s} u_{s}(t) \tag{G1-15}
\end{equation*}
$$

where:

$$
\begin{aligned}
& x_{s}(t): n \text { dimensional state vector, and output vector } \\
& u_{s}(t): m \text { dimensional control vector }(m \leq n) \\
& A_{s}, B_{s}: n \times n, n x m \text { and } p \times n \text { constant matrices }
\end{aligned}
$$

The model is given by:

$$
\begin{equation*}
\dot{x}_{m}(t)=A_{m} x_{m}(t)+B_{m} u_{m}(t) \tag{G1-16}
\end{equation*}
$$

where:
$x_{m}(t): n$ dimensional state vector, and output vector
$u_{m}(t): m$ dimensional control vector $(m \leq n)$
$A_{m}, B_{m}: n \times n, n x m$ and pxn constant matrices

In general, the dimension of the system's output may be different from that of the model (Ref. [G1-10], G1-7] and [G1-2]). The appropriate performance index is defined as:

$$
\begin{equation*}
J=\int_{0}^{T}\left[x_{m}(t)-x_{s}(t)^{T} Q\left[x_{m}(t)-x_{s}(t)+u^{T}(t) R u(t)\right] d t\right. \tag{G1-17}
\end{equation*}
$$

where:

$$
\begin{aligned}
& \mathrm{Q}: \mathrm{nxn} \text { positive-semi definıte matrix } \\
& \mathrm{R}: \mathrm{vxv} \text { positive-definite matrix }
\end{aligned}
$$

The performance index $J$ is to be minimized with respect to the control $u(t)$. This problem can be treated as tracking problems or servo-problems. (Ref.[G1-8], G1-3], [G1-10]). There are basically two different approaches to the problem. One is established by Ref. [G1-6]; in that work algebraic conditions for perfect following are derived. The second is developed by Ref. [G1-1], and[G1-9] and others. The development in[Ref. G1-1] is basically the same as in Ref. G1-9] with the exception that the input to the model is a zero input.

When given the system (G1-15) and the model (GI-16), and suppose that $u_{m}$ can be produced by ordinary differential equations such that:

$$
\begin{equation*}
\dot{u}_{m}=D u_{m} \tag{G1-18}
\end{equation*}
$$

then one can define the enlarged state vector $x(t)$ :

$$
x(t)=\left[\begin{array}{c}
x_{p}(t) \\
\cdots \\
x_{m}(t) \\
\cdots
\end{array}\right]
$$

$$
(G 1-19)
$$

The dynamic of the new system is given by:

$$
\begin{equation*}
x(t)+B u(t) \tag{G1-20}
\end{equation*}
$$

where:

$$
A=\left[\begin{array}{ccc}
A_{s}^{\prime} & 0 & 0 \\
0 & F_{m} & B_{m} \\
0 & 0 & 0
\end{array}\right]
$$

$$
B=\left[\begin{array}{l}
B_{s} \\
0 \\
0
\end{array}\right]
$$

where:
$A$ is a $(2 n+m) x(2 n+m)$ matrix
$B$ is a $(2 n+m) x$ v matrix

Considering the new system a regulator problem (Ref. [G1-1], [G1-10]), one can define the following performance index:

$$
\begin{equation*}
J=\int_{0}^{T}\left[x^{T}(t) \hat{Q} x(t)+u^{T}(t) R u(t)\right] d t \tag{Gl-21}
\end{equation*}
$$

where $R$ is a constant positive definite matrix, and $J$ is to be minimized with respect to $u(t)$. In order that $J$ is meaning full, $Q$ must be defined in appropriate ways. Since the error between the system and the model is to be minimized, the weighting matrix $\hat{Q}$ is defined as follows:

$$
\hat{Q}=\left[\begin{array}{ccc}
Q & -Q & 0  \tag{G1-22}\\
-Q & Q & 0 \\
0 & 0 & 0
\end{array}\right]
$$

It is easy to verify that with the definition Eq. (Gl-22) substituted into Eq. (G1-21), the performance index in Eq. (G1-3) is identical to that in Eq. (GI-9). Hence, the model following is formulated by defining the performance index. Using the Hamiltonian for the system, Eq. (Gl-6) and the performance index Eq. (Gl-9) and applying Pontryagin's maximum principle, one obtains the following optimal control (Ref.[G1-1],[G1-7] and others).

$$
\begin{equation*}
u^{*}(t)=-R^{-1} B_{s}^{T}\left[P_{11} x_{s}(t)+P_{12} x_{m}(t)+P_{13^{\prime}} u_{m}(t)\right] \tag{G1-23}
\end{equation*}
$$

or

$$
\begin{equation*}
u^{*}(t)=-K_{s} x_{s}(t)+K_{m} x_{m}(t)+K_{v} u_{m}(t) \tag{G1-24}
\end{equation*}
$$

where

$$
\begin{align*}
& K_{s}=R^{-1} B_{s} T_{P_{11}} \\
& K_{m}=-R^{-1} B_{s} T_{P_{12}}  \tag{G1-25}\\
& K_{v}=-R^{-1} B_{s} T_{P_{13}}
\end{align*}
$$

$P_{11}$ is the solution to ricatti equation, and is nxn symmetric matrix.

$$
\begin{equation*}
\mathrm{P}_{11}+\mathrm{P}_{11} \mathrm{~A}_{\mathrm{s}}+\mathrm{A}_{\mathrm{S}} \mathrm{~T}_{11}+\mathrm{Q}-\mathrm{P}_{11} \Lambda \mathrm{P}_{11}=0 \tag{G1-26}
\end{equation*}
$$

The solution $P_{11}$ of Eq. (Gl-26) is independent of the other two gainmatrices : $\mathrm{P}_{12}, \mathrm{P}_{13}$. These two matrices (Eq. G1-25) are the solutions to the following algebraic equations:

$$
\begin{align*}
& P_{12}+\left(A_{s}-\Lambda P_{11}\right)^{T} P_{12}+P_{12} A_{m}=Q  \tag{GI-27}\\
& P_{13}+\left(A_{s}-\Lambda P_{11}\right)^{T} P_{13}+P_{13} D=P_{12} B_{m} \tag{G1-28}
\end{align*}
$$

where

$$
\Lambda=B_{s} R^{-1} B_{s}^{T}
$$

The solution of Equation (Gl-28) depends on $D$ which is undesirable. Therefore, it usually makes $D$ equal to zero which means designing the system to step input (see for example, Ref. [G1-9]).

For linear systems with time-invariant parameters, it is very desirable to find a solution ( $P_{i j}$ matrices) which is time-invariant which means constant galns. The question which arises is if there exists the limit of the performance index $J$ as $T$ approaches infinity. It is well known that sufficient condition for this limit to exist is that the system (Eq. Gl-1) is completely controllable. Unfortunately the system (Eq. Gl-1) is uncontrollable by definition, so that the existence of the limit of (Eq. G1-3) is not guaranteed.

In Ref. G1-5 is shown that partially controlled systems can be divided into two parts, the part being independent of each other. Hence, the Ricatti equation has the well known steady state solution. Similarly as in the development of the model in the performance index in Ref. GI-I and G1-3.

Conditions are estabiished for perfect model following. In Ref. G1-3 is taken the difference between the system and the model equations (G1-15, G1-16) and substituted to that the optimal control given by Eq. (G1-25), we have

$$
\begin{gather*}
\left.\dot{\dot{x}}_{s}(t)-\dot{x}_{m}(t)\right)=\left(A_{s}-B_{s} K_{s}\right) x_{s}(t)+\left(B_{m} K_{m}-A_{m}\right) x_{m}(t) \\
+\left(B_{s} K_{v}-B_{m}\right) u_{m}(t) \tag{G1-29}
\end{gather*}
$$

If one equates:

$$
\begin{align*}
& A_{s}-B_{s} K_{s}=-B_{p} K_{m}+A_{m} \\
& B_{s} K_{v}=B_{m} \tag{G1-30}
\end{align*}
$$

Then from Eq. (G1-29)

$$
\begin{equation*}
\left(\dot{x}_{s}(t)-\dot{x}_{m}(t)\right)=\left(A_{s}-B_{s} K_{s}\right)\left(x_{s}(t)-x_{m}(t)\right) \tag{G1-31}
\end{equation*}
$$

and $K_{s}$ may be chosen to make the system less sensitive to variations of parameters. Up to this point, this condition (Eq. G1-30) for perfect following seems to be equivalent in both approaches. Conditions require that $B_{s}$ is invertible, which often is not the case.

In the case where $B_{s}$ is not invertible, Ref. [G1-7] canonical forms (for derivation, see Ref. G1-12) In order to solve (Eq. G1-30).

It has been said that for any pair $\left(A_{s}, B_{s}\right)$ there exists a transformation T such that:

$$
\begin{align*}
& \left.T^{-1} A_{S}^{T}=\left[\begin{array}{ccc}
0 & 1 & T \\
- & 1 & - \\
A & 1 & B
\end{array}\right]\right\} \mathrm{m}-\mathrm{m} \\
& \mathrm{~m}  \tag{G1-31}\\
& \left.\mathrm{~T}^{-1} \mathrm{~B}=\left[\begin{array}{c}
0 \\
-- \\
\mathrm{T}
\end{array}\right]\right\} \mathrm{m}, \mathrm{~m}
\end{align*}
$$

where 0 is null matrix. However, such transformation does not exist in general (see Ref. G1-13). Hence, Eq. (Gl-30) does not have a solution for every system and model, and this reduces the number of cases which can be solved.

In Ref. (G1-3) is derived the algebraic solution for model following; following that derivation for model following a command input and using the previous notation, one obtans:

$$
x_{m}(t)=A_{m} x(t)+\hat{B} u_{c}(t)
$$

Where $u_{c}(t)$ is command input and $\widehat{B}$ as a constant command control matrix. For perfect matching:

$$
x_{s}(t)=x_{m}(t)
$$

and also its first derivatives should be equal. This yields the control law:

$$
u(t)=\left(B^{+}\right)\left(A_{m}-A_{s}\right) x(t)+B^{+} \hat{B} u_{c}(t)
$$

The first derivatives of $x_{s}(t)$ and $x_{\text {m }}(t)$ will be equal for all $x(t)$ and $u_{c}(t)$ if

$$
\left[B B^{+}-I\right]\left[A_{m}-A_{s}\right] x(t)=\left[B B^{+}-I\right] \hat{B} u_{c}(t)
$$

which yields the conditions

$$
\begin{align*}
& {\left[B_{s} B_{s}^{+}-I\right]\left(A_{m}-A_{s}\right)=0} \\
& {\left[B_{s} B_{s}^{+}-I\right] \hat{B}=0} \tag{G1-32}
\end{align*}
$$

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