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LONG-TERM CF6 ENGINE PERFORMANCE DETERIORATION - EVALUATION OF ENGINE S/N 451-380

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16. Abstract						
number 451-380 at the General Ele Western Airlines engine was a mul Number N902WA to participate in t was conducted inbound prior to no an inbound test, a test following test after leading edge rework an consisted of detailed disassembly the as received deteriorated hard	ectric Aviation Se Itiple build engin the NASA-Lewis Tas ormal overhaul/ref g cleaning of the nd cleaning the st y inspection measu dware.	rvice Operation, Ont e, most recently ron k IV Program. The urbishment. The per low pressure turbine age one fan blades. rements and airfoil	cario, California noved from DC-10 investigative test formance testing a airfoils, and a The analytical surface finish o	a. This aircraft st program g included A final teardown checks of		
Included in this report is a deta analytical teardown report with an analytical assessment of the p conditions to losses in both sfc	Included in this report is a detailed analysis of the test cell performance data, a complete analytical teardown report with a detailed description of all observed hardware distress, and an analytical assessment of the performance loss (deterioration) relating measured hardware conditions to losses in both sfc (specific fuel consumption) and EGT (exhaust gas temperature).					
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1.0 INTRODUCTION

CF6-6D serial number 451-380 was selected to be the second of the Task IV (Long Term Performance Deterioration) engines in accordance with the requirements of the NASA-Lewis CF6 Jet Engine Diagnostics Program, Contract No. NAS3-20631. The rationale and justification for selection of this particular engine was submitted on June 17, 1977.

Background information had been supplied to NASA in two separate documents. "Potential Task IV Engines" compared 451-380 cruise trend data versus data for other Task IV engine candidates. The document also explained why a CF6-6D, rather than a CF6-50, engine was selected as the second Task IV engine. "Task IV Engine 451-380" presented detailed data on 451-380. This included module historical records, acceptance test data from the last outbound test cell run (at Ontario), and an update of the 451-380 cruise trend data.

The engine test plan was submitted on June 14, 1977. Included in that document was a list of test objectives, a description of the basic CF6-6 engine, an itemized test plan schedule, detailed instructions for the designated performance tests, analytical teardown, refurbishment and reassembly instructions, and an instrumentation and facilities description for the General Electric Aviation Service Operation (ASO) Ontario, California, CF6 test cell.

The instrumentation plan for this engine was submitted on June 14, 1977. This document described the instrumentation required for the performance testing of the engine. Standard airline instrumentation was requested and used to measure test cell engine performance. Additional LP turbine inlet pressure probes (P49) and HP compressor discharge temperatures (T₃ rake) were used to ensure data consistency and accuracy.

This report summarizes all of the pertinent data generated during the course of the test plan, together with an analytical assessment of the performance losses, as shown in Section 7.0.

2.0 OBJECTIVES

In accordance with the requirements outlined in the NASA-Lewis CF6 Jet Engine Diagnostic Program, Contract No. NAS3-20631, the following objectives were considered paramount for engine S/N 451-380, as one of the participants in the Task IV effort:

- Component Analyses of long-term performance deterioration with regard to deterioration magnitude and apportionment to individual components.
 - High Pressure (HP) Compressor Efficiency

- High Pressure (HP) Turbine Efficiency

- Parasitics

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- Low Pressure (LP) System Efficiency

- Thrust at Fan Speed

- Evaluation of LP turbine (LPT) performance restoration with regard to LPT vane and blade surface finish.
- Evaluation of fan performance restoration with regard to blade leading edge quality and airfoil surface cleanliness.
- Analysis of HP core losses (HP compressor, HP turbine, and parasitics) for use in correlating analytical teardown inspection results
- Obtain data for the CF6 deterioration model in terms of both component and overall (EGT and sfc) performance.

3.0 ENGINE HISTORY

CF6-6D Serial No. 451-380 was delivered new to Western Airlines (WAL) on DC-10-10 aircraft (A/C) No. N902WA-1 (left wing location) on June 14, 1973 (see "Engine Historical Record," Table I). During the period of revenue service up to installation on A/C N901WA-3 (right wing location) on May 24, 1976, the engine composition had undergone a number of changes necessitated by distress, updating and/or convenience of the operator.

As is common practice at the airline shops during overhaul, modules are interchanged between engines to facilitate a particular engine's buildup cycle. Consequently, after the last shop visit, 451-380 was composed of modules from several different engines. Table II is a listing of -380's engine module units (EMU), including the source engine, and the hours (TSN) and cycles (CSN) since new at the time of this teardown. There had been some compressor damage in the past history of 451-380, so some of the compressor blades and vanes had been replaced. The combustor liner (G44) was replaced during the last build, as were the high pressure turbine (HPT) blades and vanes and the turbine midframe (TMF) liner. The low pressure turbine (LPT) had not been serviced since it was purchased as a spare module. The Stage 1 fan blades were originally from ESN 451-436 and, of particular interest, was the fact that the leading edges had never been recontoured.

This build of 451-380 was per the "Phase III Upgrade Program," which included a number of performance and ruggedization improvements. The primary modifications included elliptical grinding of the HPT shrouds, shimming of the Stage 1 HPT nozzle, and increased cooling holes in the Stage 2 HP turbine nozzle support.

The engine was removed from the aircraft on June 19, 1977, to participate in the NASA-Lewis Diagnostics Program as the second Task IV engine. At that time, it had accumulated 12,189 hours (TSN) and 3741 cycles (CSN) since new. This included 4594 hours (TSO) and 1270 cycles (CSO) since the last overhaul. The engine was delivered to the General Electric Aircraft Service Operation (ASO) located in Ontario, California, on June 20, 1977.

Reason for Removal	Inlet Gearbox Horizontal Duplex Bearing Spalled	Combustor Liner Crack	Vibrations in HP Turbine Rotor	Turbine Midframe Liner Failure	Stage 10 Blade Failure	Removed for NASA - Task IV	
CSI	570	491	152	760	498	1270	
TSI	1300	1582	391	2521	1800	4595	
CSN	570	1061	1213	1973	2471	3741	
TSN	1300	2882	3273	5794	7594	12189	
Removed	10/16/73	6/6/74	8/1/74	7/15/75	2/26/76	6/19/77	
Installed	6/14/73	1/2/74	6/21/74	10/12/74	9/14/75	5/24/76	
Aircraft No.	902WA-1	904WA-3	904WA-1	901WA-2	905WA-1	901WA-3	

Table I. ESN 451-380 Historical Record.

	······································		
EMU	Source	TSN	CSN
Fan Rotor	451-380	12189	3741
- Stage 1 Blades	-436	11408	3462
Fan Stator	-380	12189	3741
Fan Frame	-380	12189	3741
Midshaft	-380	12189	3741
Compressor Rotor	-380	12189	3741
Compressor Front Stator	-380	12189	3741
Compressor Rear Stator	-380	12189	3741
Compressor Rear Frame	-402	10618	3157
Combustor	-388	10363	3028
- Liner	New	4594	1270
HPT Stage 1 Nozzle	-330	12435	4554
- Vanes	New	4594	1270
HPT Stage 2 Nozzle	-384	12597	3777
- Vanes	New	4594	1270
HPT Rotor	-384	12597	3777
- Blades	New	4594	1270
Turbine Midframe	-491	7167	2034
- Liner	New	4594	1270
LPT Stage 1 Nozzle	-491	7167	2034
LPT Stator	Spare	7717	2179
LPT Rotor	Spare	7717	2179
Turbine Rear Frame	-393	12125	3576
Inlet Gearbox	-380	12189	3741
Transfer Gearbox	-385	12111	3484
Accessory Gearbox	-380	12189	3741

Table II. ESN 451-380 Modules.

4.0 SUMMARY OF EVENTS

Work Order No. 181650 was prepared and issued to the shop by ASO/Ontario to accomplish the requirements of the Task IV Test Plan. The program objectives were met in the following sequence of events:

Date	Event
6/19/77	Engine S/N 451-380 was removed from WAL Aircraft S/N 901-3 at Los Angeles. Prepared for shipment to ASO/Ontario.
6/20/77	Engine arrived in Ontario late second shift. Incoming inspec- tions performed and slave hardware installed for test cell run. Suspected faulty EGT harness (upper right) accepted for the test.
6/22/77	Engine delivered to the test cell at 1630 hours. Test per- sonnel setting up and instrumenting per the Test Plan.
6/23/77	Initiated test at 0900 hours. Difficulties with the cell fuel system necessitated an additional calibration run.
6/24/77	Made another calibration run to substantiate the data obtained during the previous day's testing. Engine returned to Hanger No. 1 at the end of second shift for analytical inspection and refurbishment of the low pressure turbine (LPT) section.
7/1/77	LPT EMU removed and delivered to Hanger No. 2.
7/5/77	Initiated LPT disassembly and inspection checks.
7/18/77	Completed inspections and SWECO cleaning of LPT blades and vanes.
7/28/77	Completed engine rebuild. Waiting for test cell to become available.
8/2/77	Engine installed in cell. Conducted the calibration runs prior to the end of second shift. Computer nonoperational throughout the tests.
8/3/77	Computer printout showed faulty T3 readings and the fuel flow ratio was out-of-limits. Corrections made and performance runs repeated. Spent remainder of day trouble shooting fuel system (ratio between main and verification sensors still out-

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of-limits).

- 8/4/77 Most of the day spent on fuel system. Completed rerun of test at 2300 hours. Initiated removal of Stage 1 fan blades for recontouring and cleaning, per Test Plan.
- 8/6/77 Fan blades completed and reinstalled. Test runs completed at 2300 hours. Replaced suspected faulty EGT harness (upper right).
- 8/7/77 Completed testing with replacement harness at 0900. No effect on EGT indication. Engine removed from cell and delivered to Hanger No. 1 for core analytical teardown.
- 8/11/77 Initiated engine disassembly into major EMU's.
- 8/12/77 Core delivered to Hanger No. 2.
- 8/21/77 Initiated core analytical teardown.
- 8/29/77 Completed core engine inspections.
- 8/31/77 Decision made to rebuild engine at WAL. Initiated removal of HPC blades and vanes for SWECO cleaning. Also, removal of Stage 1 fan blades for fluorescent penetrant inspection (FPI) per S/B 72-653. Preparing all other hardware for return to WAL "as-is".

5.0 451-380 PERFORMANCE SUMMARY

Performance testing of engine 451-380 consisted of four separate doublepower calibrations, including an inbound test, a test following cleaning of the LP turbine airfoils, a test following Stage 1 fan blade cleaning and leading edge rework, and a final outbound test following partial refurbishment at Western Airlines. The data were reasonably consistent and repeatable, with the exception of a shift in core engine (HP compressor, HP turbine, and parasitics) performance following cleaning of the LP turbine airfoils. Detailed inspections during the core engine teardown revealed no evidence of any recent hardware distress which could have caused the performance shift.

The inbound test consisted of four down-power calibrations. A section of the EGT harness (upper right) tested bad (grounded) prior to running the inbound test, but was not replaced until the completion of the NASA program. Changing this segment and retesting (following the fan test) resulted in a negligible impact on the measured EGT. The inbound calculated T5X minus indicated EGT (Δ T5X) was +28° F, as compared to +25° F for the previous outbound test conducted at ASO/Ontario on April 30, 1976. This excellent agreement confirmed that the grounded harness segment had little effect on the indicated EGT readout.

The inbound performance level (EGT) was in excellent agreement with that projected by the cruise trend data. The total sea level deterioration since the last engine build (4594 hours TSI) was 37° C EGT and 3.2 percent sfc. The cruise data projected a value of 35° C using an 80 percent sea level to altitude EGT deterioration correlation. The inbound data are summarized in Table III.

The following component deterioration (inbound versus April 30, 1976 outbound) was measured. Note that almost all the loss was in the core engine (HP compressor, HP turbine, and parasitics):

-0.7%	AETAC (HP Compressor Efficiency)
-2.6%	∆ETAT (HP Turbine Efficiency)
+1.1%	Δ PARAS (Core Engine Internal Leakages and Cooling Flows)
-0.8%	∆ETALPS (LP System Efficiency)
+0.7%	FN at N1 (Thrust at Fan Speed)
+67° F	EGT at N1 (EGT at Fan Speed)
+71° F	T5X at N1 (Calculated EGT at Fan Speed)
+3.2%	sfc at FN (sfc at Thrust)

The performance stackup is presented using the measured inbound A4 of 54.045 square inches. The 1976 outbound test data were reduced assuming an A4 of 52.843 square inches since the HP turbine Stage 1 nozzle area was not measured at that time. The value is the average of the CF6-6 Shop Manual

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Average of Takeoff and Maximum Continuous Data.

	Inbound	After LPT	Before Fan	After Fan	Refurbished Outbound							
Date Asfc Margin AT/O EGT AM/C EGT AT/O T5X AM/C T5X %AETAC %AETAT %APARAS T5X %AFN1 %AETAC %AETAT %APARAS EGT %AETALPS %AFN1 %AA4	6/23/77 -3.2% +67° F +68° F +71° F +70° F -0.7% -2.8% -1.1% -0.8% +0.7% -2.4% -1.1% -0.7% +0.7% +2.3%	8/3/77 -4.1% +80° F +79° F +98° F +92° F -1.3% -3.6% -1.3% -0.5% +0.5% -1.3% -2.7% -1.2% -0.1% +0.5% +2.3%	8/4/77 -4.3% +93° F +88° F +109° F +99° F -1.4% -4.4% -1.7% -0.3% +0.6% -1.4% -3.5% -1.6% +0.1% +0.6% +2.3%	8/6/77 -3.8% +95° F +88° F +110° F +103° F -1.3% -4.4% -1.7% +0.3% +1.5% -1.3% -3.4% -1.6% +0.8% +1.5% +2.3%	11/2/77 -1.4% +40° F +40° F +41° F +40° F +0.2% -1.9% -0.5% -0.1% +1.5% +0.2% -1.8% -0.6% -0.1% +1.5% 0							
Note: All parameters tabulated versus the April 30, 1976 outbound level.												

limits (Section 72-51-00). This assumed value is probably too small, since the calculated value for parasitics decreased by 1.1 percent, while, in reality, this value should either stay constant or increase with time.

The test following cleaning of the LP turbine airfoils consisted of several power calibrations. The core engine (HPC, HPT, and parasitics) performance level deteriorated during the testing sequence. As noted earlier, the analytical teardown hardware inspections revealed no indication of any recent hardware distress. The magnitude of the core deterioration (+26° F EGT and +1.1 percent sfc) concealed any measured LPT improvement due to the airfoil cleaning. The calculated LP system efficiency, however, indicated there may have been as much as a 0.5 percent improvement in the LP turbine.

In addition to the core engine shift, there was a problem with an erratic facility fuel flow measurement (main to verification ratio). This problem appeared to be position-related, as the main (upstream) fuel meter always indicated a higher reading than the verification (downstream) meter. A complete investigation of the fuel flow system resulted in the erratic nature of the fuel flow ratio being eliminated. The main fuel flow, although within limits, continued to read approximately 0.3 percent higher. The performance data are summarized in Table III. The two columns of past LP turbine data (Table III) are the first and last power calibrations recorded during this testing sequence. The following component differences were measured relative to the inbound test (average of EGT and T5X):

-0.6% ΔΕΤΑC -0.5% ΔΕΤΑΤ -0.2% ΔΡΑRAS +0.5% ΔΕΤΑLPS -0.2% FN at N1 +13° F EGT at N1 +27° F T5X at N1 +0.9% sfc at FN

Two power calibrations were run following the Stage 1 fan blade cleaning and leading edge recontouring. The back-to-back test data indicated a significant improvement in both sfc (0.5% sfc at FN) and thrust (+0.9% FN at N1). There was no improvement in EGT at N1, since both fan airflow and efficiency increased. The performance data are summarized in Table III. The following component differences were determined from the back-to-back tests involving the fan blade refurbishment:

+0.1% ΛΕΤΑC 0 ΔΕΤΑΤ 0 ΔΡΑRAS +0.7% ΔΕΤΑLPS +0.9% FN at N1 +2° F EGT at N1 +1° F T5X at N1 -0.5% sfc at FN

Following completion of the NASA program workscope, Western Airlines (WAL) elected to perform a minimum workscope to make 451-380 a serviceable engine. Included in this workscope was SWECO cleaning of the HP compressor airfoils in order to demonstrate the potential gains resulting from this cleaning process.

The rebuild of 451-380 took place at WAL and included replacement Stage 1 fan blades, the original HP compressor with SWECO cleaned airfoils, a new rotating forward CDP seal, a serviceable combustor, mini-nozzle, HP turbine Stage 1 nozzle module, and HP turbine module, and the original LP turbine module. The outbound performance test indicated a 55° F EGT and 2.4 percent sfc improvement over the prerefurbished levels. Included was a 1.5 percent increase in measured HPC efficiency resulting from SWECO cleaning of the HPC airfoils. The performance data are summarized in Table III.

6.0 POSTTEST TEARDOWN RESULTS

Following the testing on 8/7/77, engine 451-380 was returned to the shop for an analytical teardown inspection of the core engine. Earlier in the program, the low pressure system had likewise been subjected to an analytical teardown. The results of the various inspection checks are presented in this section. An assessment of the hardware with regard to performance losses appears in Section 7.0. The inspection results include observations concerning the hardware which were performance-related, and do not imply that no other discrepancies existed.

6.1 HIGH PRESSURE COMPRESSOR SECTION

6.1.1 HP Compressor Rotor Assembly

6.1.1.1 General

Except for the normal accumulation of dirt on the blades and spool, the high pressure compressor rotor assembly was in very good condition. Blades in Stages 15 and 16 were very rusty in appearance, probably due to being outside the hanger during a very heavy rain storm, prior to the teardown. The rain was also responsible for the powdery substance noted throughout the compressor, on blades and spool parts, as well as on vanes and cases. Figure 1 is a photograph of the rotor. (Note: There was no evidence of any aluminum deposits. The splatter observed on the blades in the photograph is the powdery substance mentioned above.)

6.1.1.2 Rotor Land Rubs

There were no vane-to-spool rubs on any rotor land. However, there was a heavy scratch on the Stage 4 land, more than likely caused by an end vane being out of position when the cases were removed (a common disassembly fault).

6.1.1.3 Rotor Land Coating Condition

Inspection of the aluminum coating on the compressor rotor lands revealed discrepancies as noted below. (See Figure 1.)

Stage	Land Condition
14 and 15	100 percent missing
12 and 13	Approximately 50 percent missing
10 and 11	Intermittent chipping, aft end of land
7 – 9	Intermittent chipping, forward end of land
3 - 6	Satisfactory



Figure 1. HP Compressor Rotor Coating.

6.1.1.4 Blade Airfoil Condition

The airfoils were in excellent condition, except for the dirt and the oxidized appearance of the Stage 15 and 16 blades. The blades contained no nicks (FOD) or other damage. Stage 15 blades exhibited a very mild tip rub which was substantiated by the rub noted on the compressor stator land, lower case (See Section 6.1.2.4).

The Stage 1 blade midspan shrouds were visually inspected, as assembled, and were found to be in excellent condition.

6.1.1.5 Blade Surface Finish

Two blades per stage were removed from the rotor to measure the airfoil surface finish by means of a profilometer supplied by Airline Support Engineering, Evendale. The measurements were taken at 15, 50 and 85 percent of blade height at:

1. 10 to 15 percent of chord from leading edge on the suction side.

2. 10 to 15 percent of chord from trailing edge on the pressure side. Results (RMS μ inch) are presented in Table IV.

6.1.1.6 Rotating CDP Seal

A visual inspection of the CDP seal showed it to be in excellent condition. However, measurements revealed the teeth diameters to be slightly under the shop manual serviceable limits. It had already been planned to replace this seal with an updated part prior to the inspection checks.

Results of the measurements of each of the CDP seal teeth are shown in Table V. (Note: 12 o'clock position arbitrarily chosen to establish a base point and all measurements are relative to that point. Runout data are recorded in mils and are positive, unless otherwise indicated. Diameters are recorded in inches.)

6.1.2 High Pressure Compressor Stator Assembly

6.1.2.1 General

Inspection of the HP compressor stator assembly showed it to be in very good condition, except for the normal dirt buildup on the vanes. A mixture of oil and dirt was noted on the IGV's and the IGV inner shroud. There was no trace of aluminum splatter on any of the vanes. Photographs of the stator assembly, Figures 2 and 3, give the appearance of aluminum splatter, but this is some type of powdery substance presumed to be caused by allowing the engine to sit outside during a heavy rain storm. Table IV. HP Compressor Rotor Airfoil Surface Finish Inspection Results.

Stage	Overall Avg.		37		38		35		39		39		49		41		47		43		42		48		47	C L)	23		54	·	20		
	Stg. Avg.		61		48		42		43		43		55		44		48	-	44		44		51		45	С У	2	54		55		55		nes)
	Avg.	59	64	50	47	40	45	44	42	44	42	48	62	42	46	46	50	43	45	44	77	52	49	38	53	00		- 28	49	62	49	62		β μ incl
Concave	Root	44	55	43	33	29	38	38	40	48	45	53	83	42	53	50	58	64	57	40	43 4	63	62	45	58	с С С С		202	65	60	63	06	r.	= 15 RM
	Pitch	70	70	35	38	27	30	38	39	38	39	53	55	45	42	42	43	25	42	40	47	50	48	36	09	0 2 2	376	() 09	40	57	48	45		en new =
	Tip	63	66	72	70	63	67	55	46	47	42	37	48	38	44	46	48	41	35	51	42	43	38	34	40	ר 4 ר ר) r (77	42	68	35	51		rage wh
	Stg. Avg.		12		28		28		36		35		43		38		46		43		38		46		49	07	, t	52		52		45		s. (Ave
	Avg.	12	12	26	31	29	28	36	36	34	36	41	46	36	41	46	47	45	40	37	38	46	46	51	46	50 7 7	n o t ư	45	48	56	45	44		inches
Convez	Root	15	10	25	33	33	27	48	40	4 6	50	51	60	44	60	54	60	58	50	40	48	61	60	65	09	С ч С ч		55	54	09	55	50		5 RMS
	Pitch	12	13	30	30	27	29	29	37	30	33	37	42	37	32	40	4.2	45	40	42	42	42	46	47	45	4/	57	47	51	55	40	46		nish = 2
	Tip	10	13	23	30	27	27	30	30	27	25	35	35	27	30	43	38	33	30	30	25	36	32	42	33	4/	57	34	38	53	40	37		face fi
	Position No.	₽	ς		2	ß	9	Ω	4	ŝ	9	9	5	ŝ	9	ц	9	5	9	S	9	ŝ	9	Ŋ	9	о u) IU	9	Ŀ	9	4		e blade sur
	Stage			2		e		4		5		9		7		ω		6		10		11		12		т Т	14	r 1	15		16			Average

	1	r			1	·
Runout	D	E	F	G	H	I
12 o'clock 1 o'clock 2 o'clock 3 o'clock 4 o'clock 5 o'clock 6 o'clock 7 o'clock 8 o'clock 9 o'clock 10 o'clock	$\begin{array}{c} 0.0 \\ -0.5 \\ -0.5 \\ -0.5 \\ -0.5 \\ 0.0 \\ 0.5 \\ 0.0 \\ 0.5 \\ 0.5 \\ 0.5 \\ 0.5 \\ 0.5 \end{array}$	$\begin{array}{c} 0.0 \\ -0.5 \\ -0.5 \\ -0.5 \\ -0.5 \\ 0.0 \\ 0.0 \\ 0.0 \\ 0.5 \\ 0.5 \\ 0.0 \end{array}$	$\begin{array}{c} 0.0\\ 0.0\\ -0.5\\ -0.5\\ -0.5\\ 0.0\\ 0.5\\ 0.5\\ 0.5\\ 0.5\\ 1.0\\ \end{array}$	$\begin{array}{c} 0.0\\ 0.0\\ 0.0\\ 0.0\\ 0.0\\ 0.5\\ 0.5\\ 1.0\\ 0.5\\ 0.5\\ 0.5\\ 0.5\\ 0.5\\ 0.5\\ 0.5\\ 0$	$\begin{array}{c} 0.0\\ 0.5\\ -0.5\\ -0.5\\ -0.5\\ -1.5\\ -1.0\\ 0.5\\ 1.0\\ 1.0\\ 0.5 \end{array}$	$\begin{array}{c} 0.0\\ 0.0\\ 0.0\\ 1.0\\ 0.5\\ 1.0\\ 0.0\\ 1.0\\ 0.0\\ 1.5\\ 1.0 \end{array}$
11 o'clock	0.0	0.0	1.0	0.5	-0.5	0.0
Diameter at 12 o'clock FIR	17.130 0.001	17.329 0.001	17.525 0.0015	17.729 0.001	17.926 0.002	18.122 0.0015
	Diamet	ers were	Calculate	d to be:		
Maximum Minimum Average	17.130 17.129 17.129	17.329 17.3285 17.329	17.5255 17.5245 17.525	17.7295 17.729 17.729	17.928 17.925 17.927	18.1245 18.122 18.123
	Sh	op Manual	Dimensio	ns		
Maximum Minimum Serv. Limit	17.134 17.132 17.129	17.334 17.332 17.329	17.534 17.532 17.529	17.734 17.732 17.729	17.934 17.932 17.929	18.134 18.132 18.129

Table V. CDP Seal Tooth Inspection Results.

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Figure 2. HP Compressor Stator Case, Upper Land Rubs.



Figure 3. HP Compressor Stator Case, Lower 15th Stage Rub.

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6.1.2.2 Condition of Variable Stator Bushings

Variable stator bushings were in excellent condition. Inspection checks revealed no loose vanes, with all appearing to have retained their normal torque.

6.1.2.3 Vane Airfoil Condition

Except for the dirt, the airfoils were in excellent condition. No nicks, dents or other discrepancies were noted on any vane.

6.1.2.4 Casing Rubs

Inspection of the lower stator case lands revealed a heavily rubbed area on Stage 15 at approximately the 8 o'clock position. The rub was only about 3/4 inch long and the full width of the blades, but appeared to have no depth, indicating that it must have been a high spot in the coating. No rubs were seen in any of the other stages in the lower case.

Inspection of the upper stator case lands revealed some very light blade tip-to-case rubs in Stages 1 through 11. The rubs were generally the full width of the blade and were confined, in most instances, to the area of the shadow-masking. Maximum depth of any of the rubs was approximately 0.003 inch. Figures 2 and 3 are detailed photographs depicting the case land rubs.

6.1.2.5 Vane Surface Finish

Two vanes per stage, Stage 7 through OGV's, were removed from the stator case to inspect the airfoil surface finish. Measurements were taken with a profilometer at 15, 50 and 85 percent of vane height at:

- 1. 10 to 15 percent of chord from the leading edge on the convex side.
- 10 to 15 percent of chord from the trailing edge on the concave side.

Results (RMS μ inch) are presented in Table VI.

6.1.3 Compressor Rear Frame

6.1.3.1 General

A cursory inspection of the compressor rear frame (CRF) revealed no notable discrepancies. A visual inspection of the combustor and dimensional inspections of the CDP seal and the No. 4B pressure balance seal (mini-nozzle) were made. The results are presented in the following paragraphs. HP Compressor Stator Vane Surface Finish Inspection Results. Table VI.

		ŀ																					
Stage	Overall Avg.		83		56		80		65		59		52		. 61		67		72		09		
	Stage Avg.		102		58		77		70		70		50	-	68		70		72	-	42	les new	
	•Avg.	110	93	58	58	73	80	67	73	72	68	63	37	67	70	72	68	78	65	47	37	of var	
ncave	Root	70	75	40	~ 09	75	75	70	80	70	75	.09	50	10	75	65	70	75	65	50	40	finish	es)
C	Pitch	80	75	65	55	80	75	75	80	65	75	50	35	65	60	70	65	80	70	35	30	surface	S µ inch
	Tip	180	130	70	09	65	90	55	60	80	55	80	25	65	75	80	70	80	09	55	40	verage	15 RM
	Stage Avg.		63		54		83		59		48		53		54		64		72		78	hes. (A	11
	Avg.	62	63	53	55	87	78	65	53	50	45	58	48	58	50	63	65	72	72	73	83	μ inc	
Convex	Root	55	55	45	60	90	80	55	40	50	50	70	65	09	45	60	55	70	75	65	80	66 RMS	
	Pitch	70	65	55	50	80	80	80	55	40	35	45	30	60	55	65	65	75	65	75	85	Vanes =	
	Tip	60	70	60	55	06	75	60	65	60	50	60	50	55	50	65	75	70	75	80	85	lish of	
	Position No.	37	72	18	34	38	74	41	80	41	80	41	80	41	80	45	88	45	88	56	110	Surface Fir	•
	Stage	7		8		6	-	10		11		12		13		14	-	15		OGV		Average	

6.1.3.2 Combustor

A visual inspection of the combustor was made as-installed in the compressor rear frame. The most notable observation was the distress seen in the inner liner, which was the same condition noted in the previous Task IV engine, S/N 451-479; i.e., cracks originating at the aft thimble louvers. From the louver located at approximately 6 o'clock, a crack progressed aft approximately 2-1/4 inches, then circumferentially about 3/4 inch. Another crack, originating at the same louver, progressed circumferentially approximately 1-1/2 inches, then forward about 9/16 inch.

The louvers at 11 and 12 o'clock had circumferential cracks on each side ranging from 3/4 to 1 inch in length. Other louvers were similarly cracked, but to a lesser degree. Figure 4 shows a photograph of the combustor depicting the various cracks.

The outer liner had its visual axial cracks at the aft end, the longest being approximately 2 inches. At the 6 o'clock position, there was a 6 inch circumferential crack through the band cooling holes in the forward end of the ring, immediately aft of the ring containing the thimble louvers. At the same axial location, there was a 1 inch crack through the cooling holes at the 1 o'clock circumferential position.

6.1.3.3 Stationary CDP Seal, Forward

Measurements of the stationary seal lands were inadvertently missed; therefore, the analytical assessment of losses for the forward CDP seal (see Section 7.0) is based entirely on the amount that the rotating seal was under nominal blueprint requirements (see Section 6.1.1.6). Although it is possible that there was some loss due to the stationary seal, none could be assessed. Normally, this would be a minimal amount, if any.

6.1.3.4 No. 4B Pressure Balance Seal

A visual inspection of the No. 4B pressure balance seal (mini-nozzle) revealed cracks at the aft inner edge of two vanes, each approximately 1 inch in length. Since the cracks had not opened, they would have no influence on performance.

Eight equally spaced diameters were measured and recorded for each land of each of the aft seals contained in the mini-nozzle. These seals mate with the seals on the forward shaft of the high pressure turbine rotor and are referred to as the aft CDP seal (shaft forward seal) and the HPT balance piston seal (shaft aft seal).

The dimensional inspection checks are recorded in Table VII and VIII. Seal clearances are contained in Section 6.2.3.6, "Forward Shaft Seals." All dimensions are in inches.







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	в70													
Diameter	Fl	F2	F3	F4*	F5	F6								
1 2 3 4 5 6 7 8	7.948 7.942 7.943 7.947 7.948 7.947 7.947 7.947 7.946	8.115 8.111 8.111 8.114 8.112 8.112 8.112 8.112 8.114	8.602 8.591 8.595 8.596 8.596 8.598 8.600 8.599	8.756 8.751 8.755 8.754 8.754 8.753 8.755 8.755										
Minimum Maximum Average	Minimum Maximum7.942 7.9488.111 8.2708.448 8.4488.591 8.6028.751 8.756Average7.9468.1138.276 8.2758.452 8.4528.597 8.5978.754													
	Shop Manual Dimensional Requirements													
Minimum Maximum Serv. Limit	im 7.942 8.102 8.262 8.422 8.582 8.742 im 7.945 8.105 8.265 8.425 8.585 8.745 it 7.947 8.107 8.267 8.427 8.587 8.747													
* Note: In comparing the individual seal lands with the nominal Shop Manual Dimensional Requirements, land F4 shows a marked difference from that seen on the other lands. Therefore, F4 was omitted from the overall average that was used for the analytical assessments of performance loss for the seal in Section 7.0.														

Table VII. No. 4B Pressure Balance Seal, Forward Seal (Aft CDP) Dimensional Inspection.

			B71									
Diameter	F1	F2 *	F3	F4	F5	F6						
1 2 3 4 5 6 7 8	10.482 10.469 10.475 10.475 10.471 10.471 10.471 10.471	$10.616 \\ 10.602 \\ 10.603 \\ 10.608 \\ 10.615 \\ 10.613 \\ 10.609 \\ 10.606$	10.802 10.793 10.799 10.799 10.798 10.799 10.800 10.800	10.971 10.961 10.963 10.962 10.962 10.962 10.963 10.962	11.127 11.121 11.123 11.120 11.121 11.122 11.125 11.125	11.275 11.270 11.273 11.272 11.272 11.271 11.270 11.270						
Minimum Maximum Average	10.469 10.482 10.474	10.602 10.616 10.609	10.793 10.802 10.799	10.961 10.971 10.963	11.121 11.127 11.123	11.270 11.275 11.272						
	Shop	Manual D:	imensional	l Requirem	nents							
Minimum Maximum Serv. Limit 10.442 10.446 10.448 10.602 10.606 10.608 10.762 10.766 10.768 10.922 10.926 10.926 11.082 11.086 11.086 11.242 11.246 Maximum Serv. Limit 10.448 10.608 10.768 10.928 11.088 11.248												
* Note: Land F2 omitted from the average used for the analytical assessment of performance loss for this seal (see Section 7.0). F2 versus shop manual dimensions is quite different from the other lands on the seal.												

Table V	III.	No.	4B	Pressure	Balance	Seal,	Aft	Sea1	(HPT	Balance
		Pist	:on)	Dimensio	onal Insp	pection	n.			

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6.2 HIGH PRESSURE TURBINE SECTION

6.2.1 Stage 1 High Pressure Turbine Nozzle Assembly

6.2.1.1 General

Visual inspection of the Stage 1 HPTN assembly showed it to be in a less deteriorated condition than the assembly from the first Task IV engine (NASA-CR-135381, "Long-Term CF6 Engine Performance Deterioration - Evaluation of Engine 451-479"). Photographs of the assembly are shown in Figures 5 and 6. No burning of the vane trailing edge was in evidence, which was in sharp contrast to what was seen on the previous engine, and, although there were 10 vanes which were bowed on the trailing edge, none were as severe as those seen on the 451-479. As before, the bowing caused radial cracks just forward of the aft cooling holes on the concave side. The lengths of the cracks were proportional to the severity of the bowing.

There were the normal minute cracks on the leading edge of the vanes, together with the common splatter buildup, but all cooling holes were open. Five vanes were mildly burned on the leading edge, but they had not burned through. One vane, vane No. 20, did burn through, leaving a hole approximately 7/16 inch x 1/4 inch. Its insert was exposed, but was intact with no signs of burning.

The concave side of the vane also had the normal splatter, causing them to be quite rough. The convex side was smooth, which is typical. The surface finish of the airfoils was measured on three vanes and the results are tabulated in Section 6.2.1.5, "Airfoil Surface Finish Checks."

The thermal shield located inside the vane platform ID was slightly warped, but, compared to 451-479, it was in very good condition.

The aft face of the Stage 1 vane outer hook showed 360° contact. However, similar to 451-479, the first vane of each segment (CW, ALF) appeared to be the heavier marred of the two vanes.

One outer fishmouth seal tab was burned to the point of not being useful, while four others had some degree of insignificant burning. Only one inner seal tab showed any signs of burning, and it was only about 10 percent. The tab discrepancies have no influence on performance and are mentioned only as casual observations.

6.2.1.2 Drop Dimension - CRF to Stage 1 Vanes

Drop dimensions from the compressor rear frame (CRF) aft flange to the aft face of the Stage 1 vane outer hook were taken at 16 equally spaced locations, starting at 12 o'clock and working clockwise. Upon completion, it was noted that only 15 readings had been recorded. The minor differences in the readings that were recorded did not warrant a repeat of all the measurements. The drop dimensions recorded were as follows:



Figure 5. Stage 1 HPTN Vanes, Leading Edge.

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Figure 6. Stage 1 HPTN Vanes, Trailing Edge.

1.	4.855	9.	4.860
2.	4.848	10.	4.857
3.	4.846	11.	4.859
4.	4.852	12.	4.861
5.	4.860	13.	4.857
6.	4.856	14.	4.848
7.	4.855	15.	4.856
8.	4.853	16.	

Average = 4.855 inches

Note: Stackup of the mating parts is contained in Section 6.2.2.5, "Nozzle Support."

6.2.1.3 Vane Outer Platform Gap Measurements

The gaps between the outer platforms on adjacent vane segments were measured at 16 places (every other segment) at the aft end of the vanes and were as follows:

1.	0.035	9.	0.004
2.	0.012	10.	0.013
3.	0.018	11.	0.014
4.	0.009	12.	0.015
5.	0.016	13.	0.035
6.	0.003	14.	0.030
7.	0.010	15.	0.000
8.	0.012	16.	0.014

Average Gap = 0.015

Shop Manual Limits = 0.015/0.045

Note: All readings are in inches.

6.2.1.4 Stage 1 HPTN Area Check (A4)

Area measurements of each of the individual vanes were made using Tool 2C6505 and the total area (A4) calculated as shown in Table IX. When using this tool, a correction factor of 0.366 must be added to the sum of the individual vane measurements to produce the actual area of the assembly.

Nozzle No.	Area	Nozzle No.	Area	Nozzle No.	Area	Nozzle No.	Area		
$ \begin{array}{c} 1\\ 2\\ 3\\ 4\\ 5\\ 6\\ 7\\ 8\\ 9\\ 10\\ 11\\ 12\\ 13\\ 14\\ 15\\ 16\\ \end{array} $	0.833 0.897 0.844 0.834 0.875 0.850 0.928 0.864 0.850 0.817 0.855 0.902 0.858 0.808 0.797	$ \begin{array}{r} 17 \\ 18 \\ 19 \\ 20 \\ 21 \\ 22 \\ 23 \\ 24 \\ 25 \\ 26 \\ 27 \\ 28 \\ 29 \\ 30 \\ 31 \\ 32 \\ \end{array} $	0.891 0.787 0.828 0.828 0.864 0.838 0.847 0.882 0.815 0.830 0.806 0.817 0.780 0.866 0.799 0.788	33 34 35 36 37 38 39 40 41 42 43 44 45 46 47 48	0.835 0.837 0.841 0.833 0.840 0.905 0.833 0.794 0.846 0.840 0.851 0.821 0.821 0.880 0.874 0.838 0.794	49 50 51 52 53 54 55 56 57 58 59 60 61 62 63 64	0.812 0.824 0.824 0.872 0.842 0.820 0.844 0.851 0.773 0.763 0.852 0.878 0.814 0.843 0.827 0.815		
Total = 53.681 square inches Corr. Factor = 0.366 Actual A4 = 54.045 square inches Shop Manual Limits = 52.313/53.373 square inches.									

Table IX. Stage 1 HPT Nozzle Area Measurements.

6.2.1.5 Airfoil Surface Finish Checks

Three vane segments were removed from the assembly to check the airfoil surface finish. Measurements were taken at the pitchline at 10, 50 and 90 percent chord, each side. Following are the results of these measurements (RMS μ inch):

			Convex		Concave			
Vane No.	Fwd	Mid	Aft	Avg	Fwd	Mid	Aft	Avg
1	85	30	60	58	120	130	650	300
2	100	50	45	65	80	85	200	122
3	100	65	60	75	120	290	600	337
Overal1	Average	2				253		

6.2.2 <u>Stage 2 High Pressure Turbine Nozzle Assembly</u>

6.2.2.1 General

The Stage 2 high pressure turbine nozzle assembly was in relatively good condition, considering the amount of running time on the parts. The results of an overall visual inspection are discussed in the following paragraphs. Related photographs are shown in Figures 7 through 10.

6.2.2.2 Shroud Rubs and Condition of Bradelloy

The Stage 1 shrouds were very rough due to oxidation/erosion. The majority of the shrouds revealed local hot spots with a considerable amount of Bradelloy missing between the pins which were recessed, probably due to pin oxidation.

The Stage 2 shrouds were slightly eroded with a moderate rub at the 4 o'clock position (see Figure 8.)

6.2.2.3 Vane Condition

Overall, the vanes were in good condition. There were six vanes with minor leading edge burning, but none burned through. Six other vanes exhibited small cracks just below the outer platforms on the trailing edge.

6.2.2.4 Filter Screen

The filter screen had no discrepancies, other than being extremely dirty.

6.2.2.5 Nozzle Support

A visual inspection of the forward flange that supports the Stage 1 HP turbine nozzle vanes revealed contact throughout the full 360° circumference. However, in many instances, the first vane of each segment (CW,ALF) produced heavier markings than the other. All of the support cooling holes were open.

Drop checks from the forward face of the aft flange to the forward face of the flange that supports the Stage 1 vane outer hook (Dimension "K" in the Shop Manual) were taken at 16 equally spaced locations. Following are the results of these measurements (all readings are in inches):

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Figure 7. Stage 1 HP Turbine Shroud.



Figure 8. Stage 2 HP Turbine Nozzle Assembly.

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Figure 9. Stage 2 HP Turbine Nozzle Vane, Leading Edge Damage.



Figure 10. Stage 2 HP Turbine Nozzle Vane, Trailing Edge Crack.

1.	4.853	9.	4.845
2.	4.854	10.	4.846
3.	4.851	11.	4.851
4.	4.847	12.	4.853
5.	4.851	13.	4.852
6.	4.852	14.	4.853
7.	4.849	15.	4.851
8.	4.839	16.	4.850
	Average	4.850	
	Shop Dimensions	4.857/4.8	61
	Serviceable Limits	4.853/4.8	65

Corresponding dimensions from the CRF aft flange to the aft face of the Stage 1 vane outer hook averaged 4.855 in. (see 6.2.1.2 "Drop Dimensions - CRF to Stage 1 Vanes"). The shim that mounts between the nozzle support/CRF flanges measured 0.022 inch average. Therefore, the gap between the Stage 1 outer hook and the support's forward flange calculates to be 0.027 inch average. When the shims were originally introduced, it was the intent of the design to produce a gap of 0.042 inch to an interference of 0.012 inch.

6.2.2.6 Interstage Seal Grooves

The grooves worn into the interstage seal lands were measured at four equally spaced positions for each land. The results are shown in Table X.

	1		2	2		3	4		
Location	Width	Depth	Width	Depth	Width	Depth	Width	Depth	
12 o'clock	0.100	0.055	0.100	0.040	0.103	0.048	0.090	0.030	
3 o'clock	0.085	0.057	0.102	0.060	0.105	0.054	0.097	0.045	
6 o'clock	0.100	0.044	0.100	0.050	0.102	0.037	0.075	0.032	
9 o'clock	0.110	0.050	0.110	0.070	0.105	0.057	0.095	0.030	
Average	0.099	0.052	0.103	0.055	0.104	0.049	0.089	0.034	
		.1							

Table X. Stage 2 HPT Nozzle Interstage Seal Groove Measurements.

6.2.2.7 Stage 1 and Stage 2 Shroud Radii

The Stage 2 high pressure turbine nozzle assembly was restrained on the shroud grind fixture and the entire combination centered on the machine. Each stage of shrouds was measured at axial locations approximately 1/2 inch from the leading edge and 1/4 inch from the trailing edge at each end, and at the center of each shroud. Measurements at each of the axial locations consisted of a diameter recording at the 12 o'clock position and runouts relative to this point at each of the other positions (see Tables XI and XII).

6.2.3 High Pressure Turbine Rotor Assembly

6.2.3.1 General

Except for the Stage 1 blades, a visual inspection of the high pressure turbine rotor assembly revealed it to be in good condition. Most of the Stage 1 blades exhibited tip cracks and burning, as depicted in Figure 11. The concave surface of the airfoils revealed heavy deposits and roughness, while the convex sides were smooth.

The Stage 2 blade airfoils were very smooth on both surfaces. A heavy rub, at some time, was indicated by the slight discoloration noted at the blade tips, convex side, together with very mild burrs across the full blade width.

The visual inspection revealed no discrepancies in any of the spool parts (disks, shafts, seals, etc.). However, dimensional checks showed some hardware to be beyond the serviceable limits. These inspection checks are recorded later in this report.

The Stage 1 forward and Stage 2 aft blade retainer wires were removed (typically, in very small pieces) and returned to Evendale. Inspection of the pieces revealed good contact between the seals and the blades. In fact, after the retainers were removed, the seals were, for the most part, stuck to the blades and disk and had to be pried off.

6.2.3.2 HP Turbine Rotor Airfoil Surface Finish

Three blades were removed from each stage and measurements of the airfoil surface finish were obtained. Measurements were taken on each side at 10, 50 and 90 percent of the blade chord. The results, (RMS μ inch) are shown in Table XIII.

		. <u> </u>			- <u></u>					
	1/2 in.	From Leadi	ng Edge	1/4 in. From Trailing Edge						
Shroud No.	· 1	2	3	1	2	3				
$ \begin{array}{c} 1\\2\\3\\4\\5\\6\\7\\8\\9\\10\\11\\12\\13\\14\\15\\16\\17\\18\\19\\20\\21\\22\\23\\24\end{array} $	$ \begin{array}{r} -2 \\ 8 \\ 14 \\ 13 \\ 11 \\ 7 \\ 8 \\ 18 \\ 14 \\ 9 \\ 5 \\ 9 \\ 11 \\ 9 \\ 14 \\ 12 \\ 13 \\ 14 \\ 5 \\ 20 \\ 10 \\ 10 \\ 10 \\ 10 \\ 3 \\ 3 \end{array} $	$\begin{array}{c} 0\\ 11\\ 13\\ 11\\ 14\\ 11\\ 12\\ 17\\ 13\\ 10\\ 7\\ 13\\ 10\\ 7\\ 13\\ 11\\ 11\\ 18\\ 16\\ 15\\ 15\\ 15\\ 15\\ 17\\ 18\\ 15\\ 6\\ 8\\ 7\\ \end{array}$	$ \begin{array}{r} 3 \\ 14 \\ 15 \\ 10 \\ 12 \\ 8 \\ 13 \\ 20 \\ 11 \\ 7 \\ 12 \\ 13 \\ 11 \\ 16 \\ 15 \\ 8 \\ 17 \\ 11 \\ 18 \\ 16 \\ 11 \\ 9 \\ 2 \\ 6 \\ \end{array} $	$ \begin{array}{c} 3 \\ 6 \\ 13 \\ 11 \\ 4 \\ 8 \\ 3 \\ 8 \\ 11 \\ 8 \\ 13 \\ 8 \\ 6 \\ 1 \\ 10 \\ 13 \\ 6 \\ 14 \\ 8 \\ 12 \\ 3 \\ 8 \\ 10 \\ -4 \\ \end{array} $	$\begin{array}{c} 0\\ 12\\ 16\\ 11\\ 11\\ 11\\ 8\\ (46)^{1}\\ 15\\ 9\\ 11\\ 6\\ 8\\ 9\\ 11\\ 16\\ 11\\ 10\\ 16\\ 13\\ 9\\ 11\\ 6\\ 6\\ -2 \end{array}$	$ \begin{array}{c} -2\\ 12\\ 15\\ 5\\ 8\\ 4\\ 9\\ 13\\ 9\\ 14\\ 4\\ 4\\ 8\\ 11\\ 16\\ 12\\ 8\\ 8\\ 16\\ 14\\ 11\\ 10\\ 3\\ 0 \end{array} $				
All readings above are in mils and are positive, unless otherwise indi- cated. (1) Reading taken into eroded area, not included in average.										
cated. (1) Reading taken into eroded area, not included in average.LeadingTrailingDiameter at 12 o'clock33.241 in.33.237 in.Radius at 12 o'clock16.615 in.16.614 in.Minimum Radius16.613 in.16.610 in.Maximum Radius16.635 in.16.630 in.Average Radius16.626 in.16.623 in.										

Table XI. HP Turbine Stage 1 Shroud Runout Data.

	<u>1/2 in</u>	. From Lead	ting Edge	lge 1/4 in. From Trailing		
Shroud						
No.	1	2	3	1	2	3
1	-7	0	8	-12	0	-7
2	5	2	-7	-11	-8	-15
3	-4	-3	-3	-16	-7	-13
4	-5	2	-2	-15	-9	-15
5	-2	2	-2	-14	-6	-14
6	-4	-10	-13	-16	-14	-22
7	-17	-19	-10	-25	-25	-23
8	-13	-10	-5	-26	-12	-12
9	6	-2	-3	-15	-4	-15
10	3	-8 .	0	-16	-13	-8
11	0	3.	-5	-9	-1	-11
All readi	ngs above ar	e in mils a	and are pos	itive, ur	less otherwi	se
indicated	•					
			Leading	2	Trailing	
Diameter	at 12 o'cloci	k	34.551	inches	34.554 inc	hes
Radius at	12 o'clock		17.282	inches	17.288 inc	hes
Minimum R	adius		17.263	inches	17.262 inc	hes
Maximum R	adius	inches	17,288 inc	hes		
Average R	adius		17.278	inches	17.275 inc	hes

Table XII. HP Turbine Stage 2 Shroud Runout Data.

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Figure 11. HP Turbine, Stage 1 Blade Tip Cracks and Tip Burning.

			Co	nvex		Concave			
Stage	Blade No.	FWD	MID	Aft_	Avg	Fwd	Mid	Aft	Avg
1	1	90	63	40	64	450	450	250	383
	2	75	130	35	80	190	430	110	243
	3	110	43	42	65	180	550	140	290
Ave	cage		70	306					
2	1	60	37	46	48	43	50	48	47
	2	70	28	35	44	38	45	53	45
	× 3	45	38	. 70	.51	40	37	55	44
Average				·	48	· ·		•	45

Table XIII. HPT Rotor Airfoil Surface Measurements.

6.2.3.3 Rotor Blade Tip Measurements

The HP turbine rotor was setup in a lathe bed and the blades shimmed per the shop manual. Runouts at two axial locations (0.100 in. from both the leading edge and from the trailing edge) of each blade, together with the maximum blade radius of each stage, were taken and recorded as shown in Tables XIV and XV.

6.2.3.4 HP Turbine Blade Clearances

Calculated clearances, as derived from blade tip measurements and shroud dimensions (Section 6.2.2.7), were as follows:

Stage No.	B/P (in.)	Min (in.)	Max (in.)	Avg (in.)	$\Delta B/P$ (in.)
1 (Leading Edge)	0.072	0.070	0.107	0.087	0.015
(Average)	0.072	0.069	0.109	0.087	0.015
2 (Leading Edge) (Trailing Edge)	0.075	0.074 0.063	0.101	0.095 0.081	0.020 0.006
(Average)	0.075	0.068	0.101	0.088	0.013

No.	Fwd	Aft	No.	Fwd	Aft	No.	Fwd	Aft	No.	Fwd	Aft	
$ \begin{array}{c} 1\\2\\3\\4\\5\\6\\7\\8\\9\\10\\11\\12\\13\\14\\15\\16\\17\\18\\19\\20\\21\\22\\23\\24\\25\\26\\27\end{array} $	4 4 1 5 1 4 4 1 6 2 4 1 5 1 5 2 0 5 1 8 2 7 1 9 3 9 2	$\begin{array}{c} 2 \\ 4 \\ 0 \\ 17 \\ 10 \\ 2 \\ 7 \\ 1 \\ 5 \\ 0 \\ 20 \\ 10 \\ 12 \\ 2 \\ 13 \\ 3 \\ 7 \\ 5 \\ 0 \\ 13 \\ 5 \\ 9 \\ 10 \\ 16 \\ 3 \\ 16 \\ 2 \end{array}$	$\begin{array}{c} 28\\ 29\\ 30\\ 31\\ 32\\ 33\\ 34\\ 35\\ 36\\ 37\\ 38\\ 39\\ 40\\ 41\\ 42\\ 43\\ 44\\ 45\\ 46\\ 47\\ 48\\ 49\\ 50\\ 51\\ 52\\ 53\\ 54\end{array}$	$ \begin{array}{r} 11 \\ 3 \\ 9 \\ 3 \\ 15 \\ 6 \\ 8 \\ 3 \\ 7 \\ 2 \\ 7 \\ 3 \\ 6 \\ 3 \\ 4 \\ 2 \\ 5 \\ 4 \\ 5 \\ 3 \\ 2 \\ 0 \\ 3 \\ 1 \\ 5 \\ 3 \\ 3 \\ 1 \\ 5 \\ 2 \\ 3 \\ 3 \\ 1 \\ 5 \\ 2 \\ 3 \\ 3 \\ 1 \\ 5 \\ 2 \\ 3 \\ 3 \\ 3 \\ 1 \\ 5 \\ 2 \\ 3 \\ 1 \\ 5 \\ 2 \\ 3 \\ 1 \\ 3 \\ 1 \\ 5 \\ 3 \\ 3 \\ 1 \\ 5 \\ 3 \\ 3 \\ 1 \\ 3 \\ 1 \\ 5 \\ 3 \\ 3 \\ 1 \\ 1 \\ 5 \\ 3 \\ 1 \\ 1 \\ $	9 3 12 11 17 5 14 4 10 8 14 6 10 9 8 6 14 10 9 8 6 14 10 9 12 9 7 9 7 19	55 56 57 58 59 60 61 62 63 64 65 66 67 68 69 70 71 72 73 74 75 76 77 78 79 80	3 6 2 6 2 6 2 6 3 6 2 4 3 4 2 3 3 7 3 6 4 6 2 3 2 6 3 6 2 6 3 6 2 6 2 6 2 6 2 6 2	8 8 9 8 11 12 7 10 14 11 22 9 11 7 14 10 4 19 10 3 6 10 20 7 5 2 7	82 83 84 85 86 87 88 89 90 91 92 93 94 95 96 97 98 99 100 101 102 103 104 105 106 107	9 1 7 2 10 3 9 2 8 3 6 1 5 2 4 0 6 2 7 2 3 0 5 1 6 1 5	2 1 14 4 10 3 8 9 7 16 6 3 5 9 11 3 6 2 6 5 4 11 * * *	
<pre> 27 0 5 54 9 9 81 3 7 108 5 * * - Trailing edge measurements of five blades not recorded. 0 = 16.543 inches = maximum blade radius Runouts are in mile and are possible. </pre>												
	Runouts are in mils and are negative.ForwardAftMaximum Radius (inches)16.543Minimum Radius (inches)16.528Average Radius (inches)16.53916.535											

Table XIV. Stage 1 HPTR Blade Runout Data.

No.	Fwd	Aft	No.	Fwd	Aft	No.	Fwd	Aft	No.	Fwd	Aft
$ \begin{array}{c} 1 \\ 2 \\ 3 \\ 4 \\ 5 \\ 6 \\ 7 \\ 8 \\ 9 \\ 10 \\ 11 \\ 12 \\ 13 \\ 14 \\ 15 \\ 16 \\ 17 \\ 18 \\ 19 \\ 20 \\ 21 \\ 22 \\ 23 \\ 24 \\ 25 \\ 26 \\ 27 \\ 28 \\ 29 \\ \end{array} $	$\begin{array}{c} 12\\ 12\\ 12\\ 12\\ 11\\ 11\\ 12\\ 10\\ 12\\ 11\\ 12\\ 12\\ 12\\ 16\\ 14\\ 17\\ 14\\ 15\\ 12\\ 14\\ 15\\ 12\\ 14\\ 12\\ 11\\ 13\\ 14\\ 14\\ 14\\ 14\\ 14\\ 14\\ 14\\ 14\\ 14\\ 14$	3 3 1 2 1 2 1 2 1 2 1 2 1 2 1 2 1 2 1 2	$\begin{array}{c} 30\\ 31\\ 32\\ 33\\ 34\\ 35\\ 36\\ 37\\ 38\\ 39\\ 40\\ 41\\ 42\\ 43\\ 44\\ 45\\ 46\\ 47\\ 48\\ 49\\ 50\\ 51\\ 52\\ 53\\ 54\\ 55\\ 56\\ 57\\ 58\end{array}$	$ \begin{array}{r} 14\\ 15\\ 15\\ 15\\ 15\\ 19\\ 16\\ 18\\ 17\\ 22\\ 20\\ 22\\ 17\\ 20\\ 22\\ 17\\ 20\\ 21\\ 17\\ 20\\ 17\\ 20\\ 17\\ 20\\ 17\\ 20\\ 17\\ 20\\ 17\\ 20\\ 17\\ 20\\ 17\\ 20\\ 17\\ 20\\ 17\\ 20\\ 17\\ 20\\ 17\\ 20\\ 17\\ 20\\ 17\\ 20\\ 17\\ 20\\ 17\\ 23\\ 20\\ 19\\ 18\\ 16\\ 16\\ 19\\ $	7 5 9 8 8 10 9 9 12 8 11 8 9 7 8 8 10 10 6 7 5 8 7 9 5 8 11	59 60 61 62 63 64 65 66 67 68 69 70 71 72 73 74 75 76 77 78 79 80 81 82 83 84 85 86 87	17 16 18 18 18 17 19 17 22 19 22 17 21 19 23 17 22 16 22 15 19 14 18 14 20 15 18 14 17	7 9 7 9 8 7 8 8 7 9 6 7 9 6 7 9 6 7 10 6 11 7 8 5 8 3 6 5 6 3 6 4 6	88 89 90 91 92 93 94 95 96 97 98 99 100 101 102 103 104 105 106 107 108 109 110 111 112 113 114 115 116	$\begin{array}{c} 13\\ 16\\ 13\\ 17\\ 13\\ 16\\ 13\\ 18\\ 12\\ 15\\ 12\\ 15\\ 12\\ 15\\ 12\\ 14\\ 12\\ 15\\ 13\\ 12\\ 16\\ 12\\ 12\\ 12\\ 12\\ 12\\ 12\\ 11\\ 11\\ 12\\ 12$	$ \begin{array}{r} 5 \\ 5 \\ 5 \\ 5 \\ 5 \\ 5 \\ 5 \\ 5 \\ 5 \\ 5 \\ 5 \\ 3 \\ 6 \\ 2 \\ 3 \\ 3 \\ 2 \\ 2 \\ 3 \\ 3 \\ 2 \\ 2 \\ 1 \\ 2 \\ 3 \\ 1 \\ 2 \\ 0 \\ 2 \\ 0 \\ 2 \\ 0 \\ 2 \\ 0 \\ 2 \\ 0 \\ 2 \\ 0 \\ 2 \\ 0 \\ 2 \\ 0 \\ 2 \\ 0 \\ 2 \\ 0 \\ 2 \\ 0 \\ 2 \\ 0 \\ 2 0 \\ 2 0 \\ 2 0 \\ 2 0 2 $
	0 = 17.199 inches = maximum blade radius Runouts are in mils and are negative.										
	· · · · · · · · · · · · · · · · · · ·	Maxim Minim Avera	um Rad um Rad ge Rad	ius (i ius (i ius (i	nches) nches) nches)		Forwa 17.18 17.17 17.18	<u>rd</u> 9 6 3		<u>Aft</u> 17.19 17.18 17.19	9 7 4

Table XV. Stage 2 HPTR Blade Runout Data,

It is estimated that there was an additional increase of 0.010 inch in Stage 1 clearances due to shroud erosion/oxidation. Therefore, the Stage 1 average clearance was calculated to be approximately 0.097 inch, or 0.025 inch over blueprint.

6.2.3.5 Thermal Shield Seal Teeth

While the HP turbine rotor was in the runout fixture, measurements of the thermal shield seal teeth were made. To accomplish this, a position, designated as 12 o'clock, was arbitrarily selected and marked on each tooth. The diameters were taken and recorded at these positions, together with runouts at 12 equally spaced locations relative to these positions. Results are shown in Table XVI.

6.2.3.6 Forward Shaft Seals

Measurements of the teeth of the HP turbine rotor forward shaft seals were accomplished in the same manner as the thermal shield teeth; i.e., a diameter and 12 equally spaced runouts for each tooth.

During these inspections, it was noted that the attachment of each of the seals to the shaft had not been updated to the six-pin configuration; rather, they were still the original three-pin retention design.

A review of the dimensional inspection checks revealed a large amount of circumferential warping of the aft air seal (see Figure 12 which depicts the runout of the seal teeth, H1, H3 and H6). Experience has shown that this condition is the result of thermal ratcheting caused by nonuniform heating of the seal journal, resulting in distortion of the seal teeth diameters, unequal wear, increased seal leakage and, therefore, reduced performance. To alleviate this condition, a six-pin retention system was introduced for each of the seals, but rework of existing hardware was left to the convenience of the operators.

Table XVII shows the results of the measurements of the forward shaft seals. The runouts are recorded in mils and are positive, unless otherwise indicated. Diameters are recorded in inches.

6.3 LOW PRESSURE TURBINE SECTION

As previously stated, one of the prime objectives of the NASA-Engine Diagnostics Program is to evaluate low pressure turbine (LPT) performance restoration. As done on the previous Task IV Engine, S/N 451-479, immediately after the inbound performance test, the LPT EMU was removed from the engine and disassembled into its major components for various inspection checks and for cleaning of the blades and vanes.

Runout	VI	V2	V3	ν4
12 o'clock	0.0	0.0	0 0	0.0
1	1.0	-0.5	1.5	2.0
2	2.0	1.5	2.5	3.5
3	1.0	2.0	2.0	4.0
4	1.0	1.5	3.0	4.0
5	1.0	1.0	1.5	1.5
6	0.5	-0.5	1.0	2.5
7	0.5	0.0	0.5	1.0
8	0.0	0.0	1.0	2.0
9	2.0	1.0	2.0	1.5
10	2.0	1.0	2.5	3.0
11	0.5	0.5	-0.5	1.0
Diameter at 12 o'clock	26.616	26.443	26.298	26.047
Maximum Diameter	26.619	26.446	26.303	26.052
Minimum Diameter	26.616	26.443	26.298	26.047
Average Diameter	26.618	26.445	26.300	26.049
Shop Manua	al Dimens	ional Requ	irements.	
Maximum Diameter	26.630	26.470	26.308	26.058
Minimum Diameter	26.622	26.462	26.300	26.050
Serv. Limit	26.615	26.455	26.293	26.043
	1			
Note: Runout readings a otherwise indicat	are in mi ced. Dia	ls and are meters are	e positive e in inche	, unless s.

Table XVI. HPTR Thermal Shield Seal Teeth Measurements.



Figure 12. HPTR Forward Shaft Aft Seal Teeth Runout.

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Formand Cool									
				C/	<u>C5</u>	6			
Runout	GL	62	63	64	GS .	60			
12	0.0	0.0	0.0	0.0	0.0	0.0			
1	0.5	-1.0	1.0	0.0	1.5	-0.5			
2	0.5	-0.5	0.5	1.0	1.0	-0.5			
3	1.5	0.0	-0.5	-0.5	0.0	0.0			
4	0.5	-1.0	-3.0	-1.0	1.0	1.0			
5	-0.5	-3.0	-1.0	-0.5	1.0	1.0			
6	-1.0	-1.0	0.5	0.0	2.0	0.5			
7	0.0	-1.0	0.5	0.5	1.5	0.0			
8	2.0	2.0	0.5	0.5	0.5	0.5			
9	2.5	1.0	1.0	1.0	2.0	1.0			
10	2.0	0.0	2.0	1.5	2.0	0.5			
11	1.0	-0.5	2.0	0.5	1.5	0.5			
Diameter at 12 o'cloc	k 7.901	8.080	8.242	8.403	8.563	8.720			
Maximum Diameter	7.906	8.082	8.243	8.405	8.564	8.721			
Minimum Diameter	7.901	8.078	8.241	8.403	8.562	8.719			
Average Diameter	7.904	8.080	8.242	8.404	8.563	8.720			
Shop Man	ual Dimensi	onal Requ	uirement	S					
Maximum	7.909	8.087	8.250	8.410	8.570	8.730			
Minimum	7.899	8.083	8.246	8.406	8.566	8.726			
MRL*	7.864	8.048	8.211	8.371	8.531	8.691			
*MRL = Maxim	num Repaira	ble Limit	È .						
With stationary seal of	lata from 6	.1.3.4, 0	learance	es were d	etermine	d to be:			
		<u>B70</u>	- -						
	1	2	3	4	5	6			
Maximum	0.024	0.019	0.018	0.026	0.020	0.019			
Minimum	0.018	0.015	0.014	0.022	0.014	0.015			
Average	0.021	0.017	0.016	0.024	0.017	0.017			
Overall average clear 0.010 in. nominal.	Overall average clearance = 0.017 inch. Stackup of production new hardware = 0.010 in. nominal.								
(Note: Land No. 4 omitted from overall average)									

Table XVII. HPTR Forward Shaft Seal Measurements.

Aft Seal									
Runout H1 H2 H3 H4 H5 H6									
12	0.0	0.0	0.0	0.0	0.0	0.0			
. 1	-13.0	-9.5	-9.0	-7.0	-4.0	-2.0			
2	-19.0	-18.0	-14.0	-13.0	-10.0	-6.0			
3	-7.0	-6.0	-5.0	-5.0	-4.0	-3.0			
4	3.0	-2.0	-0.5	-6.0	-5.0	-1.5			
5	-2.0	-1.0	1.0	0.5	1.5	1.0			
6	-23.0	-23.0	-17.0	-15.0	-10.0	-7.0			
7	-24.0	-22.0	-16.0	-14.0	-10,5	-6.0			
8	-3.0	-2.0	0.0	1.0	0.0	1.5			
9	5.0	5.0	6.0	5.0	5.0	3.0			
10	-2.0	-0.5	0.0	0.0	1.5	0.5			
11	-1.5	-2.0	0.5	0.0	0.5	1.0			
Diameter at 12 o'c	lock 10.3985	10.567	10.728	10.8904	11.053	11.2134			
Maximum	10.423	10.589	10.589 10.747 1.0.		11.065	11.222			
Minimum	10.385	10.559	10.720 10.884		11.049	11.212			
Average	10.407	10.577	10.736	10.896	11.057	11.217			
	Shop Manual	Dimension	al Requi	rements	<u>ا</u>	L			
Maximum	10.417	10.587	10.747	10.907	11.067	11.227			
Minimum	10.413	10.583	10.743	10.903	11.063	11.223			
MRL*	10.378	10.548	10.708	10.868	11.028	11.188			
*MR	L = Maximum R	epairable	e Limit	L					
With stationary se	al data from	6.1.3.4	learance	es were de	etermine	d to be:			
		B71							
ſ	1	2	3	4	5	6			
Maximum	0.049	0.029	0.041	0.044	0.039	0.032			
Minimum	0.023	0.007	0.023	0.028	0.028	0.024			
Average	0.034	0.016	0.031 .	0.034	0.033	0.027			
Overall average clearance = 0.031 inch. Stackup of production new hardware = 0.010 inch nominal.									

Table XVII. HPTR Forward Shaft Seal Measurements (Concluded).

When this work was completed, the module was rebuilt and installed on the engine. Another test cell run was then conducted to measure any change in engine and component performance. During the period of LPT refurbishment and subsequent testing, the rest of the engine was purposely not disturbed in order that any changes in performance levels could be attributed only to the refurbishment of the LPT blade and vane surface finish, rather than to some unrelated activity. After all testing was completed, the LPT EMU was removed from the engine and set aside. Following the core analytical teardown investigation, the module was returned to Western Airlines, together with all the other engine modules. The results of the LPT module analytical teardown and the extent of the refurbishments are discussed in the following paragraphs.

6.3.1 Turbine Midframe

6.3.1.1 General

Early CF6-6 engines were besieged with problems involving turbine midframe (TMF) liner cracks, resulting in many unscheduled removals. To improve TMF durability, the material of the liner was changed from Hastelloy X to HS188, accompanied with an increase in thickness.

ESN 451-380 was equipped with this improved design liner. A visual inspection of the TMF, following the initial test, showed it to be in excellent condition, with no distress noted either in the liner or in the rest of the frame. All welds in the liner were thoroughly inspected, with no discrepancies noted.

6.3.1.2 TMF Forward Flange (Diameter U)

The TMF forward flange outer diameter (Diameter U) serves as the primary control of concentricity of the Stage 2 HPT nozzle support, affecting HPT blade-to-shroud clearances. Diameter U was measured at the 12 o'clock position, together with runouts of the flange in relation to the Number 5 bearing. The results (inches) were as follows:

12	o'clock	0.000	6 o	'clock	- 0.006
1	¹¹ –	0.002	7	11	- 0.009
2		0.007	8	11	- 0.007
3	" –	0.006	9	11	- 0.008
4	II	0.010	10	11	- 0.009
5	" -	0.008	11	11	- 0.002

Diameter at 12 o'clock = 38.732 inches

The average diameter was calculated to be 38.726 inches. The shop manual maximum serviceable limits are 38.736/38.726 inches, with a maximum allowable FIR of 0.020 inch.

In addition to the outer diameter of the frame, the inner diameter of the liner "Z" ring also controls the concentricity of the Stage 2 nozzle support. A gage was used to check the dimension UZ (see Figure 13) throughout the full 360° circumference. Except for one snug area, approximately 2 inches long at Strut Number 7, the gage passed freely throughout.

The corresponding dimension, DS, on the Stage 2 nozzle support (refer to Figure 13) was also checked with a gage and found to be acceptable.

6.3.1.3 LPT Pressure Balance Seal

An 8 diameter measurement of the stationary LPT pressure balance seal was made, with results (inches) as follows:

	1.	19.053			5.	19.	.052	
	2.	19.052			6.	19.	.053	
	3.	19.053			7.	19.	.052	
	4.	19.053			8.	19.	.052	
Avera	ge D	iamete	r =	19.	0525	ind	ches	
Shop 1	Manu	al Req	uires		19.0	050	in./19.054	in.

Average Clearance (C27) to the rotating seal (see Section 6.3.2.2) was calculated to be 0.028 inches. Stackup of production hardware indicates a clearance of 0.031 inches nominal.

6.3.1.4 Stage 1 LPTN Vane Airfoils

The Stage 1 low pressure turbine nozzle (LPTN) assembly was removed from the TMF and disassembled. A measurement of the surface finish of the airfoils was made on the end vanes of each of two segments. The measurements were taken 0.45/0.50 inch from the leading edge (L.E.) and from the trailing edge (T.E.) each side; tip readings were taken 0.50 inch below the outer platform.

Following the inspection checks, these vanes, together with all of the other Stage 1 vanes, were SWECO cleaned for two hours (see Figure 14 which exhibits one of the vane segments before and after cleaning). The same previously inspected airfoils were then remeasured to assess the amount of performance loss attributable to airfoil surface finish changes.

The results of the measurements of the Stage 1 airfoils are grouped with those similar measurements of vane airfoils in the other stages of the LPT system in Table XVIII. Assessment of performance loss is presented in Section 7.0.



Figure 13. LPT EMU Dimensional Checks.



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	[]			Convex						Conc	ave	•
	-		Ti	Р	Pitc	h				Pit	ch	
Stage	s/n	Condi- tion	LE	TE	LE	TE	Avg.	Stg. Avg.	LE	TE	Avg	Stg. Avg.
1	THHA5549	D	160	195	115	180	163		95	165	130	
		С	140	120	110	115	121		70	115	92	
,		Δ	20	75	5	65	42		25	50	38	
1	тнна6053	D	105	120	160	85	118	141	120	185	153	142
·		C	80	80	70	60	73	97	100 ·	150	125	109
		Δ	25	40	90	25	45	44	20	35	28	33
2	B1266	D	95	100	85	90	93		105	75	90	
		С	75	65	60	60	65		75	45	60	
		Δ	20	35	25	30	28		30	30	30	
2	B0897	D	95	85	85	75	85	89	110	80	95	93
		C	75	60	65	40	60	62	75	50	60	60
		Δ	20	25	20	35	35	27	40	30	35	. 33
3	T3197	D	95	65	75	60	74		65	60	63	
l I	(· · · · · · · · · · · · · · · · · · ·	C	65	48	60	40	53		60	55	58	
			30		15	20			5	5	5	-0
3	T2245	D	105	72	100	60	84	79	90		83	73
'		C I	85	65	85	50	71	62	85	55	70	64
'			20	/ /	15	10	13	1/	5	20	13	9
4	V3973	D		12	-85	58	/2		68 50	68	68	
'			65	60	50	50	58		52	60	50	
, '	112720			112	21	8	14	70	16 7 m	8 OF	12	· -> ->
4	V3/39		13			70	84	18	/5	95	85	11
					55	6U 10	13	65 10	/U c	68	69 16	63 17
<u>-</u> '	11/25		75		22	10		1.7	2	50	10	14
2	V1425		15	90	10	02	19		80 65	50	09	
Í ' '	/		20	25	40	40	20		00	40	52	
5	V0185		65	25	50	42	29	7/	25	10	1/ 42	66
	VUIOJ			55	25	1.5 1.7	42	14	. رر ۸۰	25	02 37	.00
	1		20	رد در ا	25	28	45	40 28	40	35	27	4.) 21
Ava	Stator		20	- 23		_ 20		02	<u>ст</u>		4.5	00
Avg.			1 1	ļ . ¹				67				68
	-	Δ	i I					25	. 1			22
<u> </u>	<u> </u> J	L	<u> </u>			Ĺ	L					
Surf	Face finich	when now	= 63	บ รีกด	boc n	novim	um					

Table XVIII. LPTS Airfoil Surface Finish Inspection Results.

6.3.1.5 Rebuild

After the inspection checks and cleaning were completed, the Stage 1 LPT nozzle assembly was reassembled and installed onto the TMF. All original hardware was used on the rebuild. There were six nozzle vane segments with some material missing on the aft lips, and four others were worn. All were acceptable for the rebuild.

6.3.2 Low Pressure Turbine Rotor

6.3.2.1 General

Visually, the low pressure turbine (LPT) rotor assembly was in excellent condition. The normal dirt and roughness were noted on the blade airfoils, but there was no damage to any of them. Each blade was inspected for circumferential mating face wear, with no discrepancies noted. Spool parts were also without faults.

6.3.2.2 Dimensional Inspections

The rotor was set up in a lathe bed on the Number 6 and Number 7 journals for radii measurements of the blade tip shroud seal serrations, the air seals, and the pressure balance (P/B) seal aft tooth. (Measurement of the other P/B seal teeth was not possible with the tooling on hand). The results are presented in Table XIX.

6.3.2.3 Airfoil Surface Finish Checks (RMS)

After the dimensional inspection checks were completed, two blades from each stage were removed for measurement of the airfoil surface finish. Following these checks, the remaining blades were removed and all were SWECO cleaned for two hours. The surface finish was then rechecked on the same blades previously inspected to determine the effect of the cleaning. Assessment of the performance change due to the cleaning is discussed in Section 7.0.

Table XX presents a tabulation of the surface finish inspections; dirty (D), clean (C), and the differences (Δ). All checks were taken 0.10/0.15 inch from L.E. and T.E., each side. Tip readings were taken 0.50 inch below the blade's outer platform.

6.3.2.4 Rebuild

After the cleaning and surface finish checks were completed, the blades were reinstalled in the rotor spool in their original positions.

Blade Radii								
	FWI)	AF	Ľ	Serviceal	ole Limit		
Stage	Maximum	Minimum	Maximum	Minimum	Maximum	Minimum		
1 2 3 4 5	24.131 24.115 24.101 24.120 24.119	24.114 24.106 24.089 24.110 24.111	24.129 24.119 24.115 24.120 24.117	24.114 24.112 24.100 24.108 24.104	24.144 24.137 24.125 24.135 24.135	24.121 24.114 24.102 24.112 24.112		
		Int	erstage Sea	als Radii				
	FWI)	AF	C	Serviceable Limit			
Stage	Maximum	Minimum	Maximum	Minimum	Maximum	Minimum		
1 2 3 4 5	18.188 18.002 16.856 15.565 14.204	18.182 18.000 16.851 15.556 14.192	N, 18.008 16.861 (Miss (Miss	/A 17.997 16.850 sed) sed)	18.207 18.015 16.857 15.587 14.232	18.190 17.987 16.832 15.560 14.205		
		Press	ure Balance	e Seal Rad:	li			
Maximum Radius = 9.501 Minimum Radius = 9.496 Average Radius = 9.498 Serviceable Limits = 9.490/9.496								

Table XIX. LP Turbine Rotor Radii Measurements.

			Convex				Concave					
			Ti	Tip Pitch				Pit	ch			
Stage	S/N	Condi- tion	LE	TE	LE	TE	Avg.	Stg. Avg.	LE	TE	Avg	Stg. Avg.
1	¥3984	D	100	115	125	80	105		140	140	140	
		C	65	40	80	65	63	1	105	90	97	
		Δ	35	75	45	15	42		35	50	43	
1	V5924	D	80	120	120	85	101	103	110	135	123	132
		C	70	60	85	80	74	68	75	95	85	91
		Δ	10	60	35	5	27	35	35	40	38	41
2	B2426	D	90	90	80	75	84		80	90	85	1
		C	85	55	/0	40	63		15	80	78	
	R 0100	Δ	5	35	10	35			5	10	7 -	
2	B0193	D	/0	90	50	65	69		130	/0	100	93
		C	65	40	40	30	44	54	/0	50	60	69
	852/7	Δ	5	50	10	35	25	23	60	20	40	24
3	25747	D	60	100	50	50	65		65	50	57	
		C	60	40	4.5	40	40		60	45	52	
	7505			60	5	10	19	6	5	5	5	50
3	25685	D	80	65	50	50	61	63	60	60	60	59
		C A	70	40	45	40	49	48	40	50	45	49
	17200	Δ	10	25	5	10	12	15	20		15	10
4	X7300		6U		50 25	65		ł		55	62	
		C		43	30	20	40		10	55	57	
7	V7005		5	20	1) 70	40		66		70	2	66
4	A799J		20 20	50	10	60	11			70	10	60
			40	30	30	10	2/	22	5	10	02	6
5	V4118	D	75	80	55	10	75	- 44	70	65	67	
J	A4TT0		55	60	50	45	52		55		55	
			20	20	5	45	23		15	10	12	
5	W9292		95	85	150	95	106	91	118	110	114	91
ر ا	**) _) _	C C	65	60	95	85	76	64	90	105	97	76
		Ň	30	25	55	10	30	27	28	5	17	15
AVG	Rotor	D					80			· · · · ·	88	
****		č					55		·		69	А.
		Δ					25				19	
Surfa	ce finis	sh when	new =	45 μ	inch	es ma	aximun	1.	.			I <u></u> i

Table XX. LPTR Airfoil Surface Finish Inspection Results.

6.3.3 Low Pressure Turbine Stator Assembly

6.3.3.1 General

A visual inspection showed the LP turbine stator assembly to be in good condition. Vanes were dirty and rough, as expected, with the worse conditions in the forward stages. Rub patterns on the shrouds and the interstage seals were about the same as seen on the first Task IV engine, S/N 451-479. (see Figure 15). Castone impressions were made of the maximum depth rub pattern visually observed in each stage of shrouds and seals, all being in the lower case. A sketch of each of these is shown in Figure 16. The impressions are in the files of ASE Engineering and no further action is planned for them, unless some future activity in the program indicates a need for further study.

6.3.3.2 Airfoil Surface Finish Checks

Two vane segments on each stage were removed and the airfoil surface finish of each of the end vanes for each segment was inspected. The remaining vanes were then removed and all were SWECO cleaned for two hours. After cleaning, the previously inspected airfoils were rechecked to determine the effect of the cleaning. Assessment of the performance changes attributed to the cleaning are summarized in Section 7.0. Table XVIII is a tabulation of these surface measurements; dirty (D), clean (C), and the differences (Δ). (All measurements were taken 0.45/0.50 inch from the leading edge and from the trailing edge of each side. Tip readings were taken 0.50 inch below the outer platform.)

6.3.3.3 Rebuild

After completing the cleaning and inspection checks, the low pressure turbine stator was re-assembled using all the original hardware.

6.4 FAN SECTION

Another prime objective of the CF6 Engine Diagnostics Program is to evaluate fan performance restoration with regard to Stage 1 blade leading edge quality and airfoil surface cleanliness. Performance deterioration in the fan section is believed to be primarly attributed to changes in the fan blade leading edge due to F.O.D., erosion, etc., in addition to buildup of dirt on the airfoil.

To determine the performance effects for the fan section, the same method was used as was employed on the first Task IV engine, S/N 451-479. A test cell run, after the low pressure turbine (LPT) refurbishment, served as a baseline for subsequent tests. Following this run, the Stage 1 fan blades were removed and cleaned, and the leading edges were recontoured per the Shop

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Figure 15. LP Turbine Stator Assembly, End View of Shroud and Seal Rubs.



Manual. Another test cell run was then conducted. No other changes were made to the engine during this time period; therefore, the performance improvements achieved could be attributed to the fan blade refurbishment. Upon completion of these tests, the core analytical teardown was begun with no further work planned in the fan section.

Figure 17 shows the fan section after the test cell runs. Note the missing coating on the outer OGV's as viewed through the blades at the top. A view of these vanes from the aft side can be seen in Figure 18.

6.4.1 Fan Rotor

Upon receipt of the engine at ASO/Ontario, a visual inspection showed that 14 Stage 1 fan blades had minor nicks on the leading edges, most of which were in the blade outer panel. Erosion had produced very sharp leading edges and the airfoils were extremely dirty. Considerable time was expended in the rework and cleaning of the blades during that portion of the test plan. The blades were steam-cleaned and then hand-cleaned using a soft cloth and solvent MEK, with little effect. Finally, Scotch Brite Pads Number 7447 were used to remove the deposits and polish the blades.

Visual inspection of the Stage 2 blades revealed no discrepancies of any kind.

6.4.2 Fan Stator

The fan stator case assembly had the microballoon grooved abradable Stage 1 fan shroud material, rather than the later improvement, open faced aluminum honeycomb. The shroud was in very good condition with only a few minor nicks in the surface.

The abradable material had been removed from the midring shroud over the Stage 2 blade tips in order to eliminate the possibility of engine overpressure in the event of high unbalance. (This modification was introduced by Service Bulletin 72-647). Without the shroud, there is some minor degree of performance loss.

Much of the urethane coating on the Stage 1 fan outer OGV's was missing on either or both sides of the vanes (see Figures 17 and 18). Visual inspection revealed no other discrepancies.

6.4.3 Stage 1 Fan Blade Tip Clearances

6.4.3.1 Rotor Runout

Rotor runout was determined by measuring the clearance between each Stage 1 fan blade and the shroud at the 6 o'clock position. Measurements



Figure 17. Fan Section, Overall View - Forward Looking Aft.

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Figure 18. Stage 1 Fan Outer OGV's, Trailing Edge Coating Peeling.

were taken at both the El2 and the El3 axial locations. Tables XXI and XXII are the results of these checks:

Note: E12 location (Table XXI) is at 1.929 inches from the leading edge of the blade and E13 location (Table XXII) is at 8.099 inches, also from the leading edge of the blade.

6.4.3.2 Shroud Runout

Using the blades with the smallest clearances at locations E12 (Number 24) and E13 (Number 2), clearances between the blade tip and shroud were measured and recorded at 12 equally spaced locations, starting at 12 o'clock. The full circumference was then swept and the minimum clearance at each location was recorded. Following are the results of these measurements:

Clearances at El2 $(B/P = 0.145 \text{ inch min})$								
Position No.	Clearance	Position No.	Clearance					
12 o'clock	0.174	6 o'clock	0.171					
1	0.181	7	0.167					
2	0.174	8	0.159					
3	0.173	9	0.147					
4	0.176	10	0.145					
5 0.173 11 0.150								
Minimum Clearance = 0.142 in. at 9:30 o'clock Average Clearance = 0.166 in.								

Clearances at E13 (B/P = 1.45 inch min.)									
Position No.	Position Clearance No. Clearan								
12 o'clock	0.168	6 o'clock	0.180						
1	0.178	7	0.176						
2	0.173	8	0.168						
3	0.175	9	0.150						
4	0.178	10	0.148						
5	5 0.178 11 0.158								
Minimum Clearance = 0.145 in. at 9:30 o'clock Average Clearance = 0.169 in.									
Bla N	ide Io. (learance	Blac No	le), (Clea	rance			
----------	--	------------------------	------------	----------------	------------	--------------------------------------			
1	••••••••••••••••••••••••••••••••••••••	0.206	20		0.1	80			
2	•	0.196	21		0.1	80			
• 3	5	0.196	22		0.2	11			
4	ł	0.180	23		0.1	80			
5	i	0.195	24		0.1	71			
6)	0.182	25		0.1	86			
7	,	0.196	26		0.1	86			
8	}	0.198	27		0.1	86			
9)	0.205	28		0.1	86			
1	.0	0.191	29		0.1	90			
1	.1	0.180	30		0.1	92			
1	.2	0.185	31		0.1	95			
1	.3	0.180	32		0.1	86			
1	.4	0.190	33		0.1	90			
1	.5	0.192	34		0.1	76			
. 1	.6	0.196	35		0.1	98			
1	.7	0.180	36		0.1	95			
1	.8	0.182	37		0.1	95			
1	.9	0.180	- 38		0.1	36			
A	verage mallest	Clearance Clearance	= 2 =	0.189 0.171	in. in.	(Blade No. 24)			
А	verage	Rotor Runo	out =	0.018	in.	versus 0.014 inch maximum per B/P			

Table XXI. Stage 1 Fan Blade Runout at E12.

	Blade No.	Clearance	Blad No	e • Clearance	مر (۲۰۰۰ میلی) ۱۹۹۰ - ۲۰۰۰ میلی ۱۹۹۰ - ۲۰۰۰ ۱۹۹۰ - ۲۰۰۰ ۱۹۹۰ - ۲۰۰۰
· · · · · · · · · · · · · · · · · · ·	1	0.205	20	0.190	
	2	0.180	21	0.188	
	3	0.205	22	0.204	
	4	0.180	23	0.200	
	5	0.186	24	0.190	
	6	0.195	25	0.185	
	7	0.195	26	0.191	
	8	0.181	27	0.190	· · · · · · · · · · · · · · · · · · ·
	9	0.211	28	0.190	
	10	0.190	29	0.182	
	11	0.195	30	0.190	-
	12	0.192	31	0.205	
	13	0.190	32	0.197	
	14	0.208	33	0.197	• •
	15	0.208	34	0,180	
	16	0.200	35	0.205	
	17	0.195	36	0.195	
	18	0.190	37	0.195	
•	19	0.188	38	0.181	
	Averag Smalle	e Clearance est Clearance	e =	0.193 in. 0.180 in. (Blad	e No. 2)
	Averag	e Rotor Run	out =	0.013 in. versu maxim	s 0.014 inch num per B/P

Table XXII. Stage 1 Fan Blade Runout at El3.

6.4.3.3 Blade-to-Shroud Clearances

Using the prior measured data, the Stage 1 fan blade tip clearances were determined to be as follows:

	E12	E13	B/P
Minimum	0.142	0.145	0.145
Maximum	0.221	0.211	-
$Average^{(1)}$	0.184	0.182	0.163

(1) Average clearances are determined by adding the average shroud clearances to the average rotor runouts.

7.0 ANALYTICAL ASSESSMENT OF PERFORMANCE LOSSES

The 451-380 detailed analytical teardown inspections and measurements were evaluated, resulting in a performance stackup using influence coefficients listed in Tables XXIII and XXIV. The coefficients are based on current "best estimate" of hardware effects on engine performance and may be updated based on information learned during the NASA CF6 Engine Diagnostics Program. The performance stackup (Table XXV), relative to new engine performance, is based on the analytical teardown inspections summarized in Section 6.0. The first column (assessment) is based on the analytical measurements and influence coefficients, while the second column (measured) is based on the measured test cell performance differences between an average 1975 Evendale production engine and the Ontario inbound test (see Section 5.0).

Since its original production acceptance test in mid-1973, engine 451-380 had returned to the shop several times for major overhauling so that, at the time of its use as a Task IV engine, it was made up of a conglomeration of modules from several different engines (see Section 3.0). In addition, all of the performance improvement items identified for engine serial numbers 451-406 and up had been incorporated. Therefore, an average new CF6-6D production engine for the year 1976 (rather than 451-380 as shipped) was used as the baseline performance level.

The measured performance column values (EGT and sfc) are based on calculated EGT (T5X). Although the inbound calculated T5X minus indicated EGT (Δ T5X) is consistent with the April 30, 1976, outbound data (see Section 5.0), there is a significant difference when comparing the inbound Δ T5X to an average new production engine. The EGT readout can be influenced by hardware distress (i.e., TMF liner distortion). Therefore, T5X is a more consistent health parameter to use since it is independent of any EGT profile shifts and since both T5X and sfc are functions of fuel flow.

Note that the assessment differs from measured component analysis. A slight error in A4, T3, P3 or EPR can alter the component assessment significantly. A4, in conjunction with the measured HPT pressure ratio, is used to assess the performance tradeoff between parasitics and HPT efficiency. Furthermore, the condition of the TMF liner can effect the pressure (P49), as well as the temperature (EGT) measurement. Similarly, much effort has been placed in developing a simple, yet accurate, method of measuring compressor discharge temperature and pressure. A measurement error in either parameter will result in an incorrect component apportionment of compressor efficiency, turbine efficiency and parasitics.

The 5.14 percent sfc and 85° F EGT hardware assessments are acceptable when compared with the measured 7.1% sfc and 121° F EGT deltas. As stated earlier, the influence coefficients (Tables XXIII and XXIV) are "best estimates" which may be modified based on the results of the CF6 Diagnostics Program. In addition, the analysis obviously does not address all the

Table	XXIII.	CF6-6	Influence	Coefficients.
-------	--------	-------	-----------	---------------

Component Description P EGT T/O GR. WPT Rotor Blades Surface finish rms µ in. for 0.1% nt Stage 1 2 0.08 0.06 Stage 1 Fressure 300 2 0.08 0.06 Stage 2 Surface finish Tip clearnace for 1% nt 30 mils 21 0.85 0.62 Shrouds Surface finish Tip clearnace for 1% nt 30 mils 21 0.85 0.62 Interstage Seal 20 mils = 0.15% nt 3 0.12 0.08 0.06 Stationary Surface finish Tims µ in. for 0.1% nt 50 0.22 0.08 0.06 Stator Vanes Surface finish Tims µ in. for 0.1% nt 2 0.08 0.06 Stator Vanes Surface finish Tims µ in. for 0.1% nt 2 0.08 0.06 Stage 1 Suction 28 2 0.08 0.06 Stage 1 Suction 28 2 0.08 0.06 Both Nozzles None 18 0.72 0.54 Stage 1 Suction 28 2 0.08 0.66 Stage 1 <t< th=""><th></th><th></th><th></th><th></th></t<>				
HPT Surface finish rms µ in. for 0.1X nt Stage 1 Output Surface finish Tip clearance for 1X nt 30 mils Output 2 Output Output 2 <th< td=""><td>Component</td><td>Description</td><td>°F EGT</td><td>% sfc T/O CR.</td></th<>	Component	Description	°F EGT	% sfc T/O CR.
Rotor Blades Surface finish Tmm y in. for 0.15 nt Suction 26 µ in. 2 2 0.08 0.06 Stage 1 Suction 32 2 0.08 0.06 Shrouds Surface finish Tip clearance for 1% nt Stage 1 21 0.85 0.62 Interstage Seal 20 mils = 0.15% nt 3 0.12 0.09 Rotating Stationary Surface finish Tmm y in. for 0.1% nt Suction 20 2 0.08 0.06 Stage 1 Surface finish Tmm y in. for 0.1% nt Suction 20 0.08 0.06 0.08 0.06 Stage 1 Surface finish Tmm y in. for 0.1% nt Suction 20 2 0.08 0.06 0.08 0.06 Stage 1 Surface finish Tmm y in. for 0.1% nt Suction 28 2 0.08 0.06 0.08 0.06 Norzles None 3 3 0.12 0.08 0.06 Stage 1 Suction 28 2 0.08 0.06 0.08 0.06 Stage 2 None 3 0.18 0.18 0.18 0.18 0.18 0.18 0.18 <	HPT	· · · · · · · · · · · · · · · · · · ·		
Stage 1 rms µ in. for 0.1% nt 2 0.08 0.06 Stage 2 Suction 32 2 0.08 0.06 Stage 2 Suction 32 2 0.08 0.06 Shrouds Surface finish Tip clearance for 1% nt 30 mils 21 0.85 0.62 Interstage Seal 20 mils = 0.15% nt 3 0.12 0.09 Rotating Surface finish Tms v in. for 0.1% nt Suction 20 2 0.08 0.06 Stage 1 Suction 20 2 0.08 0.06 Both Nozzles None 1 18 0.72 0.54 Ral, Piston Seal Stage 1 3.0 11.2 0.20 0.08 0.06 Stage 1 Stage 1 3.0 12.4 0.13 0.12 0.20 0.8 0.06 Stage 1 Pressure 240 2 0.08 0.06 0.08 0.06	Rotor Blades	Surface finish		
Stage 1 Suction 20 µ in. 2 0.00 0.00 Stage 2 Suction 32 2 0.08 0.06 Strouds Surface finish 1 0.85 0.62 Shrouds 20 mils 21 0.85 0.62 Interstage Seal 20 mils = 0.15% nt 3 0.12 0.09 Rotating Surface finish 2 0.08 0.06 Statonary Surface finish 2 0.08 0.06 Statonary Surface finish 2 0.08 0.06 Stage 1 Surface finish 2 0.08 0.06 Stage 2 Pressure 140 2 0.08 0.06 Stage 1 Surface finish 2 0.08 0.06 Both Nozzles None 2 0.08 0.06 Rataring 33 mils = 0.12 WC16 to EP 18 0.72 0.54 Stage 1 Stage 2 2.0.28 n2t 1.0 0.22 0.18 Stage 1 = 0.41% n2t 2.0 0.01 0.01 Stage 1 = 0.12 m2t 3.0 0.71 0.26 Stage 1 = 0.12 m2t 2.0 0.01 0.01 Stage 1 = 0.128 m2t 2.0	Ptopo 1	rms μ in. for 0.1% nt	2	
Stage 2 Suction 32 2 0.08 0.06 Shrouds Surface finish Tip clearance for 1% nt 30 mile 21 0.85 0.62 Stage 1 20 mils = 0.15% nt 3 0.12 0.09 Notating Stationary Surface finish rus w in. for 0.1% nt 3 0.12 0.09 Stage 1 Surface finish rus w in. for 0.1% nt 2 0.08 0.06 Stage 1 Surface finish rus w in. for 0.1% nt 2 0.08 0.06 Stage 1 Surface finish rus w in. for 0.1% nt 2 0.08 0.06 Both Nozzles None 2 0.08 0.06 0.06 Both Nozzles None 3 18 0.72 0.54 Stage 1 Stage 1 Stage 1 0.02 0.06 0.06 Stage 1 Stage 1 Stage 1 0.12 w016 to HP 18 0.72 0.54 Stage 1 Stage 1 Stage 1 0.12 w016 to HP 18 0.13 w011 Stage 1 Stage 1 0.12 w016 to HP 18 0.13 w011 0.12 w016 Stage 1 0.102 w12	stage 1	Suction 26 µ in. Pressure 330	2	0.08 0.06
Shrouds Surface finish Tip clearance for 1% nt 30 mile 21 0.85 0.62 Interstage Seal 20 mile = 0.15% nt 3 0.12 0.09 Rotating Stationary Surface finish rms u in. for 0.1% nt Surge 1 3 0.12 0.09 Stater Vanes Surface finish rms u in. for 0.1% nt Surface finish resoure 240 0.08 0.06 Both Nozzles None 3 0.12 0.08 0.06 Both Nozzles None 18 0.72 0.54 Stage 1 0.1% wCl6 to HP 18 0.72 0.54 Stage 1 0.01% nt surface finish blades and vanes* 0.13 0.26 Stage 1 0.1% nt surface finish blades and vanes* 0.11 0.22 0.18 Stage 1 0.1% nt surface finish blades and vanes* 0.13 0.26 0.22 0.18 Stage 1 0.10% nt t 1.3 0.13 0.26 0.22 0.18 St	Stage 2	Suction 32	2	0.08 0.06
Stage 1 30 mile 21 0.85 0.62 Interstage Seal 20 mils = 0.15% nt 3 0.12 0.09 Rotating Stationary Surface finish 3 0.12 0.09 Stator Vanes Surface finish 7 2 0.03 0.06 Stage 1 Surface finish 7 2 0.03 0.06 Stage 2 None 2 0.08 0.06 0.08 0.06 Both Nozzles None 3 31s = 0.1% WC16 to HP 18 0.72 0.54 Ral, Piston Seal 33 mils = 0.1% WC16 to HP 18 0.72 0.54 Stage 1 Stage 2 Stage 1 0.13 0.22 0.18 Stage 1 Stage 1 0.11% n2t 3.0 0.11 0.22 0.18 Stage 1 Stage 1 0.10% n2t 1.3 0.13 0.22 0.18 Stage 2 Stage 3 0.12% n2t 1.3 0.13 0.13 0.22 Stage 1 Stage 1 0.10% n2t 1.3 0.13 0.13 0.13 0.10	Shrouds	Surface finish Tip clearance for 1% nt		
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Interstage Seals Rotating 20 mils clear Stage 1 Stage 2 stage 3 stage 4 stage 5 = 0.25% n2t = 0.14% n2t 1.8 1.0 0.10 0.19 0.10 0.09 0.7 0.07 0.07 0.07 0.07 *Pressure (concave) surface values weighted at 1/4	Stage 5	= 0.06% n2t 0.80\% n2t	$\frac{0.4}{5.7}$	$\begin{array}{ccc} 0.04 & 0.04 \\ 0.59 & 0.51 \end{array}$
Rotating 20 mils clear Stage 1 = 0.25% n2t 1.8 Stage 2 = 0.14% n2t 1.0 Stage 3 = 0.14% n2t 0.7 Stage 4 = 0.05% n2t 0.7 Stage 5 = 0.05% n2t 0.4 *Pressure (concave) surface values weighted at 1/4	Interstage Seals			
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Stage 5 $= \frac{0.05\%}{0.54\%} \frac{n2t}{n2t}$ $\frac{0.4}{3.9} \left[\begin{array}{c} 0.04 \\ 0.40 \\ 0.40 \end{array} \right] \frac{0.03}{0.34}$ *Pressure (concave) surface values weighted at 1/4	Stage 4	= 0.10% m2t	0.7	0.10 0.09
*Pressure (concave) surface values weighted at 1/4	Stage 5	$= 0.05\% \eta 2t$	0.4	0.04 0.03
*Pressure (concave) surface values weighted at 1/4		0.54% n2t	3.9	0.40 0.34
Suction (convex) curtace values weighted of 3/4	*Pressure (concave) s	urface values weighted at 1/	4	

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Table XXIV. CF6-6 Influence Coefficients.

Component	Description	° F EGT	% sfc T/O CR.
Interstage Seals Stationary Stage 1 Stage 2 Stage 3 Stage 4 Stage 5			
Bal. Piston Seal Rotating	51 mils = 0.1% WCl6 to LP from HP	2	0.25 0.2
Stationary CDP Seals			
Fwd. Seal Rotating Stationary	19 mils = 1% WC16 to HP	18	0.72 0.54
Aft Seal Rotating Stationary	33 mils = 1% WC16 to HP	18	0.72 0.54
Compressor-All Parts Rotor Blades	Dirt buildup, damage, L/E irregularity Tip Clear avg. 10 mils tighter throughout compressor = 1% nc		
	Breakdown - 10 mils each stage:		
	Blade to case: Stage 1-4 0.05% nc Stage 5-16 0.49% nc	1 9.3	0.04 0.03 0.37 0.25
	Vane to spool: Stage 3-7 0.13% nc Stage 8-15 0.33% nc Total 1.00% nc	2.5 $\frac{6.3}{19.0}$	$\begin{array}{cccc} 0.10 & 0.07 \\ \underline{0.25} & \underline{0.18} \\ 0.75 & 0.54 \end{array}$
	Surface finish: 6 rms μ in. = 0.1% ης	2	0.08 0.03
	33% of blades eroded on each stage, stage 5 on back = 0.7% nc 50% = 1.0% nc		
Compressor Casings	Leaking variable stator bushings		
Stator Vanes	Surface finish: 10 rms µ in. = 0.1%	2	0.08 0.03
Fan			
Vanes	Surface finish:		
Stage 2 OGV - Inner OGV - Outer	87 rms μ in. = 0.1% ηf 87 rms μ in. = 0.1% ηf 80 rms μ in. = 0.1% ηf	0.6 0.6 0.6	0.07 0.05 0.07 0.05 0.07 0.05
Fan Rotor Blades Stage 1	Tip clear 35 mils = 0.6% nf	3.6	0.42 0.30
Stage 2	Surface finish: 27 rms µ in. = 0.1% nf Tip clear 40 mils - 0.2% pf	0.6 1.8	0.07 0.05 0.21 0.15
	22 rms μ in. 0.1% ηf	0.6	0.07 0.05

Table XXV. Analytical Assessment of 451-380 Losses.

		Assessmen	ıt	(Based or Measur	n T5X) red
	η	EGT	sfc	T5X	sfc
HP Compressor	1.04%	20° F	0.78%	40° F	1.6%
Blade Surface Finish (30 μ in.) Vane Surface Finish (42 μ in.) Rotor Land Coating	0.50 0.42 0.12				
HP Turbine	1.39%	29°F	1 .1 8%	110° F	4.6%
Stage 1 Nozzle Surface Finish Stage 1 Blade Surface Finish Stage 2 Blade Surface Finish Stage 1 Blade Tip Clearance (+25 mils) Stage 2 Blade Tip Clearance	0.15 0.10 0.83				
(413 min) Stage 1 Shroud Roughness	0.26 0.05				
Parasitics	1.00%	18° F	0.72%	-38° F	-1.4%
Aft CDP Seal (+5 - Stationary) (+2 - Rotating)	0.15 0.06				
Balance Fiston Seal (+17 - Stationary) (+4 - Rotating) Fwd CDP Seal (+3 mils - Rotating)	0.51 0.12 0.16				
LP Systems LP Turbine	3.02%	18° F	2.46%	+9° F	2.3%
Rotor Clearance Stage 2 (1 mil) Stage 3 (1 mil)	0.01% 0.007 0.005	0	0.01%		
I/S Seal Clearance Stage 2 (5 mil) Stage 4 (19 mils) Stage 5 (27 mils)	0.23% 0.06 0.10 0.07	1.7°F	0.17%		
Blade Airfoil Surface Finish Stage 1 Stage 2 Stage 3 Stage 4 Stage 5	0.72% 0.45 0.17 0.05 0.04 0.01	5.2° F	0.53%		
Vane Airfoil Surface Finish Stage 1 Stage 2 Stage 3 Stage 4 Stage 5	0.74% 0.54 0.13 0.04 0.03 0.00	5.3° F	0.55%		
Fan Section			:		
Stage 1 Blade LE/Cleanliness Stage 1 Blade Tip Clearance	0.70%	0°	0.50%		
(20 mils) Stage 2 Shroud Removed	0.34%	2° 4°	0.24% 0.26%		
Outer OGV Peeled Coating	(hub) 0.28	0°	0.20%		
Total		85° F	5.14%	121° F	7.1%

ORIGINAL PAGE IS OF POOR QUALITY possible loss mechanisms. For instance, no method had yet been devised to completely assess the Stage 1 HPT nozzle assembly. Losses due to vane surface finish deterioration can and are assessed, but, beyond that, no influence coefficients are available to cover other detrimental conditions. Various conditions, such as ballooning, bowing or burning of vanes, in addition to the size of the gap/interference fit between the Stage 2 HPT nozzle support forward flange and the Stage 1 vane outer flange, cannot be assessed. An excessive gap would result in cooling-air leakage between the flanges, whereas an extreme interference fit would cause the vanes to be tilted forward, resulting in uneven loading, which also would allow leakage between the flanges. During the performance restoration program for 451-337 in early 1975, back-to-back engine tests were conducted to compare the original hardware versus a new Stage 1 HPT nozzle assembly. A 1.2 percent improvement in sfc was realized; however, it should be noted that the new Stage 1 HPT nozzle assembly incorporated shims to reduce vane to Stage 2 support interference and the Stage 2 HPTN support flange was reworked to 63 rms finish. Measurements during the buildup revealed a 1 mil average interference between the nozzle vane outer flange and the Stage 2 support. Effort is planned, as part of this program at a later date, to address this condition.

The Stage 1 fan blade leading edge contour and airfoil surface cleanliness is another example of hardware condition that cannot be analytically assessed as to performance loss. For this reason, the testing sequence included back-to-back tests comparing performance levels of the blades in the "as-received" condition versus performance levels of the blades after cleaning the airfoils and reworking the leading edges. A 0.5 percent improvement in sfc was demonstrated and is included in the analytical assessment of losses (Table XXV and Figure 19).

Other potential areas that do not readily lend themselves to assessment include the dirt buildup on the HPC airfoils, partial loss of the fan OGV urethane coating, and leakage paths throughout the engine (variable stator bushings, split line flanges, and piping flanges). Following consultation with Fan Aero Design Engineering, a 0.2 percent loss in sfc was assessed for the OGV coating deterioration.

The test program also included back-to-back tests comparing low pressure turbine performance with blade and vane airfoils in the "as-received" condition versus the same blades and vanes after having been cleaned by the SWECO method (see Section 6.3). The airfoil surface finish of two blades and vanes from each stage were measured both before and after cleaning, and the measurements are reported in Sections 6.3.2.3 and 6.3.3.2.

Evaluation of the airfoil surfaces prior to cleaning indicated that a performance loss equal to about 1.08% sfc could be attributed to the existing surface condition (roughness) as compared to an airfoil surface when new. After cleaning the airfoils using the vibratory mill process (SWECO), an assessment of the surface condition indicated that only about 67% of this assumed loss had apparently been recovered; i.e., about 0.7% sfc (Table XXVI). This recovery ratio is somewhat lower than the 90% improvement assessed when LPT



	1	Assessment	
LP System Refurbishment	'nη	EGT	sfc
Blade Airfoil Surface Finish Stage 1 Stage 2 Stage 3 Stage 4 Stage 5	0.44% 0.25 0.11 0.04 0.03 0.01	3.2° F	0.33%
Vane Airfoil Surface Finish Stage 1 Stage 2 Stage 3 Stage 4 Stage 5	0.50% 0.28 0.14 0.05 0.02 0.01	3.6	0.37%
Total	0.94%	7° F	0.70%

Table XXVI. Analytical Assessment of LPT Refurbished Airfoils.

airfoils of a previous service engine (S/N 451-479) were cleaned. There seems to be no apparent or obvious reason for this difference, but, it is true that the airfoils of engine 451-380 had been in service for 3249 hours longer than the 4468 hours accrued for the airfoils of engine 451-479. It is quite possible that a longer cleaning cycle may have further improved the surface quality of the airfoils for engine 451-380. It may also be that closer attention will have to be paid to the controlling variables in the cleaning process, such as the cleaning solution, so as to produce more consistent results.

The results of back-to-back tests (before and after cleaning LPT airfoils) are reported in Section 5.0 and show that there was a measured performance improvement in the low pressure system of about 0.5% sfc. This compares favorably with the assessed value of 0.7% sfc improvement as determined from surface roughness measurements obtained before and after cleaning. The results of similar back-to-back tests with engine 451-479 had indicated no measurable performance improvement. Although not conclusive, the data obtained to date indicates that some reevaluation of surface finish influence coefficients may be required.

8.0 APPENDICES

APPENDIX A - FUEL ANALYSIS

APPENDIX B - LOG SHEETS - INBOUND RUN

APPENDIX C - LOG SHEETS - REFURBISHED LOW PRESSURE SYSTEM

APPENDIX A - FUEL ANALYSIS

Following are the results of fuel analysis by the Bearings/Gears and Fuels/Lubes Lab:

Sample Identification:	1.	"Eng. No. 451-380 Inbound 6-23-77 Ontario"
	2.	"Eng. No. 451-380 after-LPT Sample No. 1, 8-2-77"
	3.	"Eng. No. 451-380 after Fan Rework Sample No. 3, 8-7-77"
Hydrogen:	1.	14.00%
	2.	13.92%
	3.	13.90%
Sulfur:	1.	0.085%
	2.	0.0115%
	3.	0.074%
Net Heat by Precision Bomb:	1.	18574 Btu/No.
	2.	18582 Btu/No.
	3.	18605 Btu/No.
Comments:	1.	Specific Gravity 0.7851 at 60° F
	2.	Specific Gravity 0.7922 at 60° F
	3.	Specific Cravity 0.7892 at 60° F

APPENDIX B - LOG SHEETS, INBOUND RUN

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APPENDIX C - LOG SHEETS, REFURBISHED LOW PRESSURE SYSTEM

	GENERAL ELECTRIC COMPANY CONTARIO, CALFORNIA AND NO Sheel of <u>CALFORNIA</u>	OUL TYPE: Mabi LAE THE OIL W. FUEL TYPE JP.4. MODEL NO. LEG. 6 D NO. OF STARTS JA	76 5147 2/5/ 3:06 35 - 3 2 LUV. WINK PARTER: /8/6-50 1011 RUNNUS THE 24 71.9	CONC. 2.2.46-2.56/60-3.5 Inneum. 12350 CONSTIM. 2.2.46-3.6 Inneum. 12350 AUGMENTED	TEMT. PREFAG. NEET PREFAG. NEET FLOW THAT THAT THAT THAT THAT THAT THAT THA	. Test gastoring check No. 2802 RUM, W ek. ROA SAN= 40 RIG, 40 - 520 ROK	· cdech - che of a cond - we - 2 - 50 eres 3 - 52 dec	20 132 122 12 1 20 0 2 2 2 7/80 - 415 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	shart 75 - 1023 - 5 Time ro 2016 35 an fired of We 2202 Ron Boe 20 Pero We 530 - 20	= 27 1316, 72 = 64 100, 20 100, 20 100 - 49 10 - 05 20 10 20 00 00 00 00 00 00 00 00 00 00 00 00	+ down Strain Roal Teleway W. 3-28 act W. W. Name	20 Core Cent dore - 0 80 238 240 - 464 0 - 0 0 0 0 0 0 0 0 0 0 0 36 3	75: 132 132 136 136 136 136 136 136 13 137 126 22 1 2369 224 100 41 251 112 600 000	$\mathbf{r} = \mathbf{r} + $	142/19821351360 60 000 80 122 832/360 - 729/2405,20 1.8.1 1 1 1 1 2 2 1 55 24 2 2	ch 72 (he h 16 a) 321 326 4.5 - 15 50 4.45 1. 3 12 .es	6.2 10 6. 50× 60× 60× 60× 60× 60× 60× 60× 60× 60× 6	21 70 23 20 2) 32.6 20 26 26 254 2.1/1/1.5 - 35/ 109 65 .7 .2 .7 .7 .5 .5 .2	24 PSI Q 9597 RPM 1620 SUMP. 3.65 , JARE: . 6 = 51.66 P31P	
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Date Barator Auchte	2-22	-	AIRCRAFT TURBINI CF 16 / CF 16	E TEST LC TF-39 TRIC COMPANY VLIFORNIA	OG SHEET	Gas Turbine Serial Na. 25.2.23.2. Nature of Test Street 22 of 260	unet e
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Date _ S	2-12	AIRCRAFT T	URBINE TEST LC	DG SHEET	Gas Turbine Secial No. 151-390	2
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INE PRESS TEMP. W	DIL TYPE: MOLAN	THE OIL WT. FUEL TYPE:	<i>b-dr</i>	AODEL NO: 2 EG- GD	NO. OF STARTS	
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Gas Turbine Serial No. 1221 - 380 Nature of Test Sheet 2 of 165 Sheets	NO. OF STARTS LAST READING NO LAST READING NO TOTAL RUNNING TIME OVER CRUISE AUGRIAL ELECTRIC ACTEDATE GENERAL ELECTRIC ACTEDATE	тали нам и поли пами и поли поли поли пами нам и поли пами пами поли поли поли поли поли поли поли пол	0C - 19 - 1 FOIN CF - 19 - 1 FOIN CF - 18 - 1 FOIN CF - 18 - 17
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Date 8-4.27	AIRCRAFT TURBINE TEST LOG SHEET GENERAL ELECTRIC COMPANY		Gas Turbine Serial No. <u>X57 - 3</u> 80 Nature of Test	
Inspector	ONTARIO, CALIFORNIA	RUN NO.	Sheel 2 of 16 Sh	Sheets
TIME PRESS TEAP. W 0 OUL TYPE. $MOQALL TF T T$ 01. TIME PRESS TEAP. W 0 START THAT OUL W. THAT 01. W. 0 NO W. 01. W. 0 NO W. 0.0 NO W.	1. WT. FUEL TYPE. J. P. 4 Молет но:	-6 77 100 1.050 100	OF STARTS ST READING NO FAL RUNNING TIME	
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A ₄	Stage One High Pressure Turbine Nozzle Area
A/C	Aircraft
ALF	Aft Looking Forward
ASE	Airline Support Engineering
ASO	Aviation Service Operation
ASO/O	Aviation Service Operation/Ontario
AVG	Average
B30	Forward Compressor Discharge Pressure Seal Clearance - Reference Shop Manual, 72-00-00
в70	Aft Compressor Discharge Pressure Seal Clearance - Reference Shop Manual, 72-00-00
B71	High Pressure Compressor Balance Piston Seal Clearance - Reference Shop Manual, 72-00-00
B/P	Blueprint
BW	Blade Width
C27	Low Pressure Turbine Pressure Balance Seal Clearance Reference Shop Manual, 72-00-00
CDP	Compressor Discharge Pressure
CR	Cruise
CRF	Compressor Rear Frame
CSI	Cycles Since Installed
CSN	Cycles Since New
CSO	Cycles Since Overhaul
CW	Clockwise
DACO	Douglas Aircraft Company
DELT	Delta
DETAC	Delta High Pressure Compressor Efficiency

SYMBOLS AND ACRONYMS (Continued)

- DETALP Delta Low Pressure System Efficiency
- DETALPS Delta Low Pressure System Efficiency
- DFN1 Delta Net Thrust at Constant Fan Speed
- DIA. Diameter
- DPARA Delta Parasitics
- DPARAS Delta Parasitics
- E12, E13 Fan Blade Tip Clearances Reference Shop Manual, 72-20-01
- EGT Exhaust Gas Temperature
- EGT Exhaust Gas Temperature Margin
- Margin
- EMU Engine Maintenance Unit
- EPR Engine Pressure Ratio
- ESN Engine Serial Number
- ETAC High Pressure Compressor Efficiency
- ETALPS Low Pressure System Efficiency
- ETAT High Pressure Turbine Efficiency
- FBW Full Blade Width
- FIR Full Indicated Runout
- FLA Forward Looking Aft
- F_N Net Thrust
- F_N @ N1 Net Thrust at Constant Fan Speed
- FOD Foreign Object Damage
- FPI Fluorescent Penetrant Inspection
- FWD Forward

SYMBOLS AND ACRONYMS (Continued)

G1,G2,G3, G4,G5,G6	High Pressure Turbine Rotor Forward Shaft, Forward Seal Teeth
H1,H2,H3 H4,H5,H6	High Pressure Turbine Rotor Forward Shaft - Aft Seal Teeth
HP	High Pressure
HPC	High Pressure Compressor
HPS	High Pressure System
HPT	High Pressure Turbine
HPTN	High Pressure Turbine Nozzle
HPTR	High Pressure Turbine Rotor
HRS	Hours
ID	Inside Diameter
IGB	Inlet Gearbox
IGV	Inlet Guide Vane
IN	Inch
I/S	Interstage
Dim "K"	Dimension "K", High Pressure Turbine Nozzle Support - Reference Shop Manual, 72-52-00
LE, L/E	Leading Edge
LP	Low Pressure
LPS	Low Pressure System
LPT	Low Pressure Turbine
LPTN	Low Pressure Turbine Nozzle
LPTR	Low Pressure Turbine Rotor
MAX	Maximum
M/C	Maximum Continuous

SYMBOLS AND ACRONYMS (Continued)

MIN	Minimum
MM	Maintenance Manual
MRL	Maximum Repairable Limit
$N1, N_1$	Fan Speed
N/A	Not Applicable
NASA- Lewis	National Aeronautics and Space Administration - Lewis Research Center
NO.	Number
No. 4B	Number Four Ball Bearing
Noz.	Nozzle
OGV	Outlet Guide Vane
P3, P3	Compressor Discharge Pressure
P ₄₉ , P49	Low Pressure Turbine Inlet Pressure
QEC	Quick Engine Connect
RAD	Radius
RMS	Root Mean Square
SB	Service Bulletin
Serve Limit	Servicable Limit
sfc	Specific Fuel Consumption
sfc MARGIN	Specific Fuel Consumption Margin
SL	Sea Level
S/M	Shop Manual
S/N	Serial Number
STG	Stage

SYMBOLS AND ACRONYMS (Concluded)

SWECO	Vibratory Mill Cleaning Process
T ₃	Compressor Discharge Total Temperature
T ₅ X	Calculated Exhaust Gas Temperature
т/с	Thermocouple
TE	Trailing Edge
TMF	Turbine Mid Frame
т/о	Takeoff
TSI	Time Since Installed
TSN	Time Since New
TSO	Time Since Overhaul
V1,V2, V3,V4	High Pressure Turbine Rotor Thermal Shield Seal Teeth
V1,V2, V3,V4 WAL	High Pressure Turbine Rotor Thermal Shield Seal Teeth Western Airlines
V1,V2, V3,V4 WAL WC16	High Pressure Turbine Rotor Thermal Shield Seal Teeth Western Airlines Sixteenth Stage Cooling Flow
V1,V2, V3,V4 WAL WC16 WF	High Pressure Turbine Rotor Thermal Shield Seal Teeth Western Airlines Sixteenth Stage Cooling Flow Fuel Flow
V1,V2, V3,V4 WAL WC16 WF Δ	High Pressure Turbine Rotor Thermal Shield Seal Teeth Western Airlines Sixteenth Stage Cooling Flow Fuel Flow Delta
V1,V2, V3,V4 WAL WC16 WF Δ	High Pressure Turbine Rotor Thermal Shield Seal Teeth Western Airlines Sixteenth Stage Cooling Flow Fuel Flow Delta Efficiency (Eta)
V1,V2, V3,V4 WAL WC16 WF Δ η	<pre>High Pressure Turbine Rotor Thermal Shield Seal Teeth Western Airlines Sixteenth Stage Cooling Flow Fuel Flow Delta Efficiency (Eta) High Pressure Compressor Efficiency</pre>
V1,V2, V3,V4 WAL WC16 WF Δ η ηc ηf	<pre>High Pressure Turbine Rotor Thermal Shield Seal Teeth Western Airlines Sixteenth Stage Cooling Flow Fuel Flow Delta Efficiency (Eta) High Pressure Compressor Efficiency Fan Efficiency</pre>
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