NASA CR-135092 BBN Report No. 3332

ANALYSIS AND DESIGN OF A HIGH TIP SPEED, LOW SOURCE NOISE AIRCRAFT FAN INCORPORATING SWEPT LEADING EDGE ROTOR AND STATOR BLADES

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NASA Contract No. NAS3-18512

December 1977

Submitted to:

NASA Lewis Research Center 21000 Brookpark Road Cleveland, Ohio 44135

1 Pages N			
1. Report No.	2. Government Accession No.	3. Recipient's Catalog	No.
NASA CR-135092			
4. Title and Subtitle		5. Report Date	
Fan Incorporating Swept Leading E	Speed, Low Source Noise Aircraft	December 1977	
and an arrange of the state of	age notor and Stator Blades	6. Performing Organiz	ration Code
7. Author(s)		8. Performing Organiz	ation Report No.
Richard E. Hayden, Donald B. Blis	s, Bruce S. Murray.		
K.L. Chandiramani, Joseph I. Smul	lin, and Pierre G. Schwaar	3332 10. Work Unit No.	
9. Performing Organization Name and Address			
Bolt Beranek and Newman Inc.		505-03-12	
50 Moulton Street		11. Contract or Grant	No.
Cambridge, Massachusetts 02138		NAS3-18512	
		13. Type of Report an	nd Period Covered
12. Sponsoring Agency Name and Address		Contract Report	1974-1977
National Aeronautics and Space Ad	ministration	14. Sponsoring Agency	Code
Langley Research Center Hampton, VA 23665			-
15. Supplementary Notes		1	
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Project Manager, James G. Lucas, NASA Lewis Research Center	V/STOL and Noise Division		
Cleveland, OH 44135			
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by rotor wakes incident on the bl	ades by progressively sweeping the for the unsteady loads along the s	vanes from root to	tip in order
structural design considerations	were required to assure the perform	mance and integrity	dynamic and
unusual blade and vane design.			, 01 01115
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dynamic, acoustic, and structural	design, and the mechanical assemb	y of the rig fan.	
7. Key Words (Suggested by Author(s))	18. Distribution Statem	ent	—. .
Acronomation, For and Communication	Noise Bedustian		
Aeroacoustics; Fan and Compressor Low Source Noise Fan; Subsonic Le	· 4		
Blades; Swept Blades; Turbofan.			
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19. Security Classif. (of this report)	20. Security Classif. (of this page)	21. No. of Pages	22. FILE
Unclassified	Unclassified	227	I

^{*} For sale by the National Technical Information Service, Springfield, Virginia 22161

FOREWORD

The purpose of this report is to describe a recently completed program to design and manufacture an experimental transonic fan model featuring novel methods for noise reduction at the source. The program was conducted between 1974 - 1976 under contract NAS3-18512 issued by NASA Lewis Research Center, with Bolt Beranek and Newman Inc. (BBN) as the prime contractor and AVCO Lycoming as a subcontractor. The contract resulted from a NASA request for proposals (RFP) concerning CTOL aircraft engine fan source reduction concepts. The intent of the RFP was to identify advanced design concepts for reducing both rotor and stator sources which could be implemented with existing aerodynamic and structural design capabilities. The RFP encouraged proposals to reduce noise from high speed single stage fans.

BBN proposed the use of "subsonic leading edge" rotor blades and variably swept stator vanes as the concepts to be investigated. The study and engineering work culminated in the fabrication of a 20-inch diameter fan stage to be tested for acoustic and aerodynamic performance at the Lewis Research Center, National Aeronautics and Space Administration, Cleveland, Ohio.

Bolt Beranek and Newman Inc. (BBN), Cambridge, Massachusetts, served as the prime contractor with overall program responsibility, as well as prime technical responsibility for the fan acoustic design, and other areas. The Lycoming Division of AVCO Corporation, Stratford, Connecticut, was a major subcontractor to BBN, with responsibilities in aerodynamic and mechanical design, and manufacture of the fan hardware. Rotor blades and stator vanes were manufactured under subcontract by New England Aircraft Products, Farmington, Connecticut.

Also included in the program were efforts to develop a 3-dimensional compressible flow computer program to analyze the flow through the rotor, especially in the vicinity of the leading edge, and the investigation of the feasibility of using porous trailing edges on the stators to reduce broadband noise. The 3-D flow program was discontinued at the time the rotor design was finalized, and the porous edge concept was not used because of the difficulties perceived in manufacture of small vanes from available porous metal materials.

Numerous individuals at BBN and AVCO made significant contributions to this project. Mr. Richard Hayden served as project

manager, and contributed to the acoustic design of the fan as well as other areas. Dr. Donald Bliss served as an associate project manager and had responsibility for the concept of the rotor blade, the rotor acoustic design, and the coordination of the aerodynamic design with AVCO. Mr. Bruce Murray also served as an associate project manager and supervised the mechanical design and manufacturing aspects of the fan. The stator acoustic design was carried out by Dr. K.L. Chandiramani, and Mr. Joseph Smullin. Drs. John McElman and John O'Callahan performed finite element stress analysis of the rotor blades, and Dr. O'Callahan contributed to numerical fluid mechanical analysis of the rotor flow field.

At AVCO Lycoming, Mr. Pierre Schwaar served as the principal investigator and has primary program responsibility for the fan aerothermodynamical design, and for implementing the subsonic rotor leading edge concept and the acoustic design of the stator blades within operational structural constraints. Mr. Herbert Kaehler led AVCO's work on structural analysis and Mr. John Banks supervised mechanical design and manufacturing activities there.

Mr. James G. Lucas of the NASA Lewis Research Center's V/STOL and Noise Division was the NASA Program Manager, and contributed valuable assistance in the mechanical design and manufacturing areas, and in the integration of the fan into NASA's test facilities.

This report has been designated as Bolt Beranek and Newman (BBN) Report No. 3332.

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SUMMARY

On current generation high bypass ratio turbofan engines, the fan is a predominant noise source which must be controlled to meet future aircraft noise goals. Of the various approaches to turbofan engine noise reduction, the most attractive is reducing the strength of the noise-producing elements at the source, thus avoiding weight and performance penalties associated with various sound suppression approaches.

In modern high bypass ratio turbofans, the fan thrust is achieved in a single fan stage, which usually requires supersonic tip speeds of the fan rotor to produce the necessary pressure rise. In such fans, the predominant sources of noise are shocks radiated from the supersonically-moving rotor blades (called multiple-pure-tone [MPT] noise), and tones radiated from the rotor wake interaction with stator vanes.

In this program, two advanced noise reduction concepts were applied to the design of a 1.6 pressure ratio single stage fan. The goal of the design was to reduce the following acoustic multiple pure tone noise, rotor-wake/stator-blade interaction noise, and noise due to operating the rotor in distorted or turbulent inflow. Unique nonradial blading of the rotor and stator was used to achieve these goals. The rotor blade leading edges were swept so that the normal component of flow to the edge is subsonic at all points along the blade span, thus preventing the occurrence of leading edge shockwaves. The stator vanes were designed to minimize noise generated by rotor wakes incident on the blades by progressivly sweeping the vanes from root to tip in order to produce subsonic trace speeds for the unsteady loads along the span. Special aerodynamic and structural design considerations were required to assure the performance and integrity of this unusual blade and vane design.

The rotor design using a blade concept with shock-free leading edges (except at points of inflection where weak conical shocks occur) is highly flexible in that a large family of blade shapes and leading edge contours may, in general, be used to achieve the noise reduction goal. The swept rotor design is also attractive since it should perform equally well at off-design conditions if it has been designed to perform properly at the highest envisioned rotor speed. The swept edges also are compatible with reducing noise generation due to inflow distortion.

In the final design of the particular rotor ultimately constructed, a reversal of the sweep direction was required near mid-span to minimize stress levels in the blade. Once this was done, aeroacoustical-structural design iterations led to a blade with acceptable stress levels and no additional compromise in acoustic performance beyond the expected weak conical shock at the sweep reversal point.

The aerodynamics of subsonic leading edge rotor cascades with supersonic absolute inflow velocities are not well known, and will clearly require further study.

The concept of forcing the trace speeds of moving load distributions on stator vanes to be subsonic was introduced for the first time in this program. The design of a stator which uses this concept requires a controlled rate of axial sweepback (or circumferential skew), the details of which depend heavily on the rotor wake field which varies with distance from the rotor. The selection of a stator vane number for a given rotor design is done with the familiar cutoff condition in mind; however, supersonic rotor tip speeds make it impossible to completely cut off the radiation at the tips of the stator vanes. No serious aerodynamic or structural problems were associated with the swept stator. The stator acoustic design procedure is now well-defined in terms of flow parameters needed as inputs, but the ability to predict the necessary flow parameters of the rotor wake field is presently limited.

SECTION 1

INTRODUCTION

With the advent of high bypass ratio turbofan engines, and the associated decrease in exhaust velocity, the fan stage has become the dominant aircraft engine noise source. Therefore, fan noise reduction is a problem of primary importance in the ongoing effort to evolve quieter aircraft. Furthermore, it is increasingly important that any penalty in operating efficiency incurred by noise reduction methods be minimized.

In general, noise reduction can be achieved in two ways: (1) reduction through the attenuation of propagating sound fields; and (2) reduction of the strength of the noise sources themselves. The first approach typically involves the use of absorptive duct liners and splitters, and possibly basic modifications to the inlet duct geometry. Because add-on features are required, and the duct length may be increased, the penalties associated with this approach are added weight and some direct reduction in aerodynamic efficiency. Furthermore, there may be a degree of noise generation associated with some treatment modifications, such as in-duct splitters, particularly if the inflow to the fan is disturbed.

The second approach, which is the reduction of noise at the source, can be pursued in many ways. The basic fan design parameters can be chosen to give more favorable acoustic behavior. For instance, the tip speed can be reduced, the spacing between the rotor and stator can be increased, and the number of blades and vanes can be altered to prevent the propagation of certain duct modes. Whether these options can be exercised in a given case depends on the design constraints on the performance and size of the system.

Because, in most circumstances, acoustic considerations cannot dictate the choice of basic fan design parameters, other means of noise source reduction are worthy of consideration. These other means of source reduction necessarily involve changes in the aerodynamic design of the blades and vanes. The design changes may occur either within the framework of conventional design practice, such as the use of optimized blade section properties, or may involve the exploration of novel concepts. Although development of all the design data needed for implementing novel concepts for noise source reduction may be initially difficult, the noise reduction potential of a successful concept may greatly exceed the reduction obtained by more conventional means. Of course, the final test of an acoustically successful concept must always

be whether any associated penalties in performance, complexity, and system integration can be overcome or, at least, justified in relation to the benefits.

The subsonic leading edge rotor is implemented by tailoring (sweeping) the rotor leading edge to the mean inflow such that subsonic Mach number flow is achieved normal to the leading edge along the entire span, thus preventing shock generation. Previous use of partially-swept transonic rotors was done in an effort to reduce transonic drag rise and thus improve stage efficiency. Swept stators have been previously used to reduce noise, but the design concept implemented here involves tailoring the leading edge shape to a detailed estimate of the rotor wake field incident upon the stator.

The remainder of this report is organized to describe in detail the rationale for selection of the particular concepts (Sections 2 and 3), details of the design procedure used on the swept rotor blades (Section 5) and stator vanes (Section 6), residual noise sources (Section 7), and facility integration (Section 8). Appendices contain a listing of aerothermodynamic design parameters (App. A), a discussion of geometric considerations for subsonic leading edge rotor blades (App. B), a detailed discussion of acoustical considerations in the stator design (App. C), discussion of empirical estimates of fan noise levels (App. D), and a useful algorithm for estimating trace speeds of rotor wakes on stator vane leading edges (App. E).

. 1

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SECTION 2

TRANSONIC FAN NOISE SOURCES

This section summarizes the major noise sources and mechanisms encounted with transonic fans. Typical design characteristics of single stage transonic fans are summarized in Table 1.

TABLE 1. TYPICAL CHARACTERISTICS OF SINGLE STAGE TRANSONIC FANS.

Pressure Ratio Range	1.4 - 1.8
Tip Speed	300 to 600 m/s (1000-2000 ft/sec)
Relative Rotor Tip Mach No.	
Rotor Inlet Hub/Tip Ratio	
Stator Hub Mach No.	.8

The most important noise sources, which involve both the rotor and stator, are:

Shockwave noise from the supersonic portion of the rotor blades, often called multiple pure tone (MPT) noise.

Rotor/stator interaction noise caused by unsteady loading due to aerodynamic interaction (tonal and broadband noise).

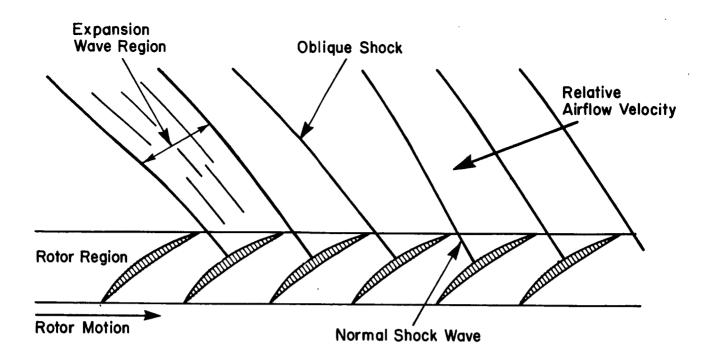
Noise caused by unsteady loading on rotor blades interacting with inflow distortions and turbulence (tonal and broadband noise).

A brief elaboration on each of these sources is now provided.

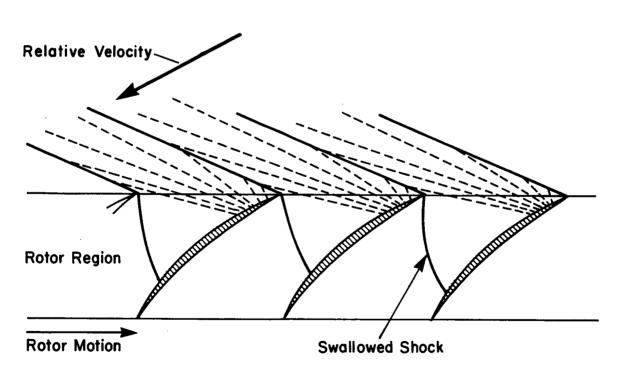
2.1 Shockwave Noise

When the relative flow past the rotor becomes supersonic, the propagation of shock waves out of the inlet duct becomes an important noise source. The upstream propagation of waves from blade rows with detached and attached shock wave patterns is shown in Fig. 1 (from Ref. 1). Because the pressure field must satisfy a periodicity condition, expansion waves occur in the regions between the shock waves.

Several investigations (Refs. 2 through 7) have shown that nonlinear effects are an important factor in the upstream shock propagation process. Because nonlinear attenuation occurs more rapidly for higher initial levels, an increase or reduction of the



(a) Detached Shock Wave Pattern



(b) Attached Shock Wave Pattern

FIG. 1. POSSIBLE SHOCK WAVE CONFIGURATIONS FOR ROTORS IN SUPERSONIC FLOW.

shock strength at the blades does not produce a comparable increase or reduction of levels at the end of the inlet duct, or in the far field. This effect is strongest when the wave train in the duct is well ordered and can be considered nearly one-dimensional in character. The important consequence of this effect is that very substantial levels of source reduction must be achieved to guarantee a worthwhile reduction in level in the far field.

Another important consequence of nonlinear propagation is the redistribution of the shock noise spectrum from blade passage frequency and its harmonics to the rotor shaft rotation frequency and its harmonics. This redistribution occurs because of blade-to-blade differences in the initial strength and position of the shock waves. These blade-to-blade differences are caused by variations in manufacturing tolerances that may affect the circumferential location, setting angle, thickness, and camber of the blades. Because the shock train structure is inherently unstable to perturbations in strength and position, these initial disturbances need not be large. As an example, when periodic variations in shock strength occur, the stronger shocks tend to overtake and dominate the weaker shocks because of nonlinear effects. Because the variations in strength are caused by blade-to-blade differences, they are periodic in the shaft rotation speed. Thus, as the wave train propagates, the harmonics of shaft speed become increasingly important relative to the harmonics of blade passage frequency. Fig. 2 shows the redistribution of energy from blade passage frequency to shaft rotation frequency as the result of an initial amplitude perturbation to one shock in a wave train. Figure 3 shows sketches of typical noise spectra for a subsonic fan, which has no shock noise, and for a supersonic fan, where the tones at the harmonics of shaft speed are clearly present. Clearly the multiple pure tone noise due to shock wave propagation is a major noise problem.

2.2 Rotor/Stator Interaction Noise

Unsteady aerodynamic loads on rotor blades or stator vanes produced by the aerodynamic interaction between the rotor and stator are an important source of both tonal and broadband noise. The main causes of the aerodynamic interaction are the interference with the potential flow pressure and velocity fields and the interaction with the viscous and turbulent wakes from upstream blades. The potential field interaction that produces tonal noise at the harmonics of the blade passage frequency can be virtually

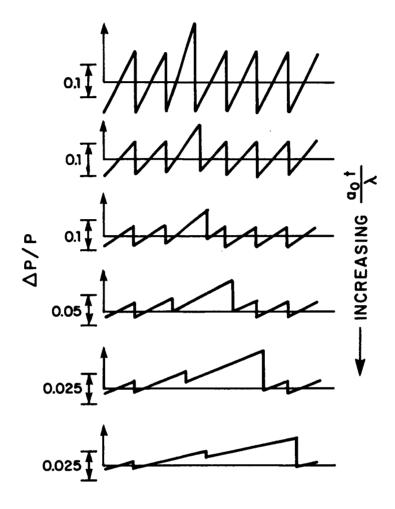
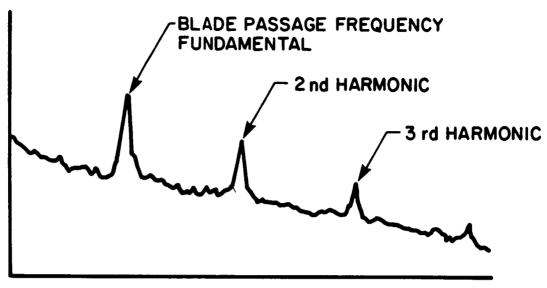


FIG. 2. DEVELOPMENT OF A SHOCK TRAIN WITH AN INITIAL DISTURBANCE (from Ref. 5).

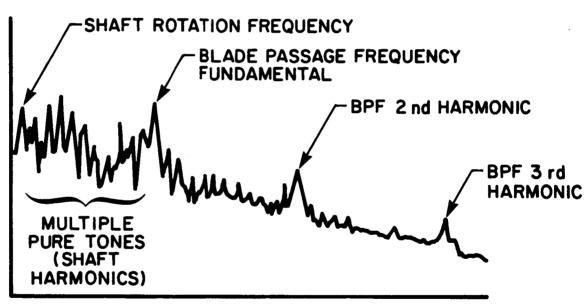




FREQUENCY

a) TYPICAL NOISE SPECTRUM FOR A SUBSONIC TIP SPEED FAN





FREQUENCY

- b) TYPICAL NOISE SPECTRUM FOR A SUPERSONIC TIP SPEED FAN
- FIG. 3. TYPICAL FAN NOISE SPECTRA FOR SUBSONIC AND SUPERSONIC TIP SPEEDS.

eliminated by providing adequate spacing between the rotor and stator. Increasing the spacing on a high by-pass ratio fan stage is usually practical and does not involve a severe aero-dynamic penalty. The interaction of the stator vanes with the "mean component" (steady velocity deficit) of the rotor wakes produces tonal noise at the harmonics of blade passage frequency, while the interaction with the wake turbulence produces broadband noise. Increasing the spacing between the rotor and stator also reduces - but does not necessarily eliminate - this noise source.

2.3 Inflow Distortion Noise

The inflow to the fan rotor typically exhibits a degree of spatial nonuniformity and a certain amount of turbulence. Sound is produced by unsteady loads on the rotor blades operating in this disturbed inflow. Steady spatial nonuniformity causes tonal noise to be produced at the harmonics of blade passage frequency, and the presence of turbulence produces broadband noise. However, if the turbulence scales are sufficiently long in the streamwise direction, then many blades will interact with a given disturbance in a similar manner, producing peaks in the noise spectrum at the harmonics of blade passage frequency. Because the basis for this noise source is a random process, the amplitude of these peaks will vary in time in a random manner. Inflow distortions have been shown to be a potentially important noise source in static fan test facilities. Their importance in an actual flight environment is less certain, since the effect of forward motion is usually to reduce certain types of inflow distortion.

SECTION 3

NOISE SOURCE REDUCTION CONCEPTS

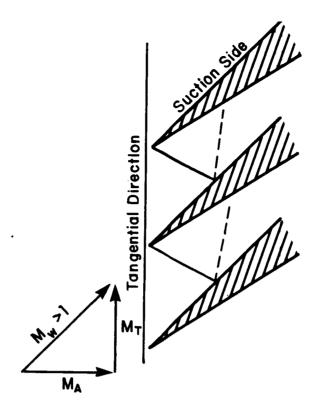
In this section, the concepts for the reduction of rotor and stator noise sources are described. A review of the detailed analysis and design procedures associated with the implementation of these concepts in the present program is postponed to the sections later in the report dealing with detailed design.

3.1 Rotor Noise Reduction

As discussed in the previous section, two noise sources associated with the rotor are multiple pure tone noise due to shock waves and inflow distortion noise. This section describes a concept which has the potential to substantially reduce multiple pure tone noise. As an additional advantage, this concept will also help reduce the problem of inflow distortion noise.

In principle, upstream-propagating shockwave noise can be reduced by designing for careful alignment of the relative velocity, w, with the suction surface near the rotor blade leading edge, as shown in Fig. 4a. However, completely shockfree entry into the blade row cannot be achieved in conventional blading because of the finite thickness of the blade leading edge. The effect of thickness is illustrated in Fig. 4b. Moreover, since the relative inflow direction varies with the operating conditions, the proper alignment cannot be maintained in off-design operation, nor in the presence of inflow distortions. Thus, this concept presents several practical difficulties for application to aircraft fans which do not operate at a single design point.

A different approach to obtain shockfree entry into a blade row is now described. It is believed that this approach does not suffer from the shortcoming of the more conventional approach just described. Consider a blade whose leading edge is swept relative to the local inflow velcocity vector. leading edge would in general appear swept when viewed from the side and skewed when viewed from the front. If the leading edge is swept such that the Mach number of the relative flow component normal to the leading edge is everywhere subsonic, a shockless leading edge results. In wing theory, this is referred to as a "subsonic leading edge in supersonic flow" (See, for instance, Ref. 8). In rotating applications, the radial variation in relative Mach number makes it possible. in principle, to completely avoid upstream shock wave propagation by using leading edge and surface generating line sweep which varies from hub to tip. In practice, structural constrains force some design compromises. In the present design, the structural

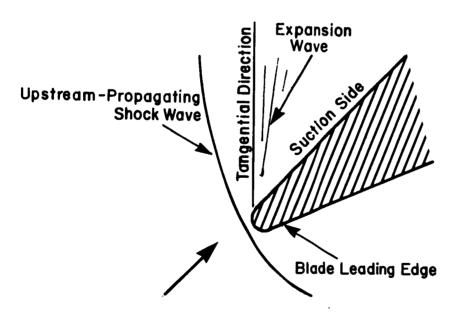


(a) Two-Dimensional Leading Edge Design Without Upstream Waves

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(b) The Effect of Leading Edge Thickness

FIG. 4. SHOCKLESS LEADING EDGE DESIGN AND THE EFFECT OF THICKNESS.

compromise entails the presence of a train of conical shocks upstream of the rotor associated with a sweep discontinuity in the leading edge. From the standpoint of preventing shock noise, the design can be made insensitive to operating conditions, relative flow alignment, and inlet distortions by designing the sweep distribution for the highest relative inflow Mach number to be expected; thus ensuring a lower subsonic normal Mach number component for off-design conditions. This insensitivity is considered to be a major asset.

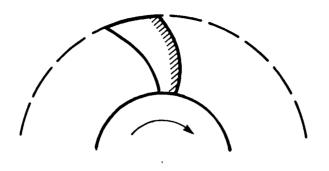
The underlying aerodynamic idea is now reviewed. shows a swept wing of infinite extent subject to an incident supersonic flow. Since there is no spanwise variation in the wing geometry, the axial component has no effect. The aerodynamic forces are determined entirely by the component of the flow normal to the wingspan. If the component normal to the span is subsonic, then there are no shock waves associated with the flow over this wing. Of course, to be completely shockless, the normal component must be sufficiently subsonic that transonic flow effects do not occur in the normal flow plane. The only effect of the axial component is in the structure of the viscous boundary layer on the wing surface, but this is not related to the presence or absence of shock waves. The same ideas are applicable, of course, to an infinite span sweptback cascade. Fig. 5b shows a finite span wing sweptback to have subsonic leading edges. The aerodynamic behavior is now considerably more complicated. In particular, the presence of conical shocks at the front and rear of the wing root and at the rear of the tips is unavoidable. These isolated points on the wing are discontinuities in the otherwise subsonic edges. The conical shocks are, however, weaker than their twodimensional counterparts and, because of their three-dimensional nature, decrease in strength with distance from their point of origin.

The application of a subsonic leading edge to a fan blade is illustrated in Fig. 5c. This illustration is simplified to its essential form, showing only the radial change in Mach number. The actual process is nonplanar because of the change in direction of the inflow with radial location. The particular case illustrated applies to a transonic fan, since part of the incident flow is subsonic. Then the leading edge can be made completely shockless even though the blade is of finite extent assuming that one is able to predict and accommodate the effects of spanwise flows (Ref. 9).

The local leading edge sweep at each radial station is chosen to be greater than the Mach angle of the local flow, i.e., the swept edge must lie within the local Mach cone. This assumes that the normal flow to the leading edge is everywhere subsonic. Because of the gradient in Mach number, the incident flow is subsonic at the base of the blade so a shock cannot emanate from this point (unlike the wing root in Fig. 3b). Hence, the blade leading edge can be entirely shockless, except for the effects of aerodynamic interference between the blade tip and the shroud which produce conical shocks. If the fan were completely supersonic, a conical shock should also occur at the root of the blade. By designing the leading edge and the other generating lines of the forward portion of the blade surface to be subsonic for the situation that produces maximum relative flow Mach number, the edge will remain subsonic under all other operating conditions. The blade leading edge would usually be designed to have a constant normal velocity (Mach number) component at all points along the span at radii (from the hub) greater than that at which the critical normal Mach number, $M_{w_{crit}}$, is reached. The critical Mach number is that normal Mach number (<1) at which thickness effects would cause the flow to become transonic.

In addition to sweepback, Figs. 6a, 6b, and 6c show swept forward and compound sweep blades that are also possible configurations. All of the blade configurations must have a conical shock at the tips caused by aerodynamic interference with the shroud. The compound sweep blade will also have a weak conical shock at the discontinuity in sweep, which is positioned somewhere along the leading edge (assuming the discontinuity lies in the region of supersonic relative inflow). Although the compound sweep blade has the acoustic penalty of introducing a weak conical shock, it offers other definite advantages. Structural considerations provide the most severe constraint to the design of high speed fans with swept blades. Fairly large excursions of the leading edge are required to implement this concept. It should be noted that the family of threedimensional curves that satisfies the subsonic leading edge condition is not unique and therefore considerable latitude exists to determine structurally optimum shapes. Figure 7 shows the type of conical shock wave pattern for a compound sweep blade. The blade in the sketch closely resembles the design developed during the course of the project being described.

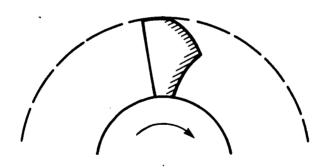
Figure 8 compares the operation of a moderately loaded blade row with and without subsonic leading edges in supersonic flow. As explained above, the subsonic edge region allows shock-free entry into the blade row. The blade rows are identical except for the addition of a subsonic leading edge region in one case. The front surface of the blade must be designed so that any shocks generated on the suction surface of



A. Swept Back



B. Swept Forward



C. Compound Sweep

FIG. 6. FRONT VIEW OF SOME POSSIBLE BLADE CONFIGURATIONS WITH SUBSONIC LEADING EDGES.

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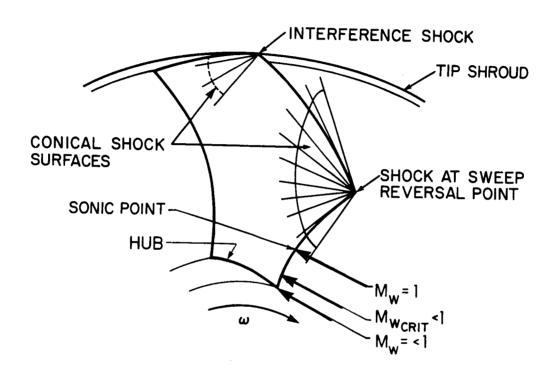
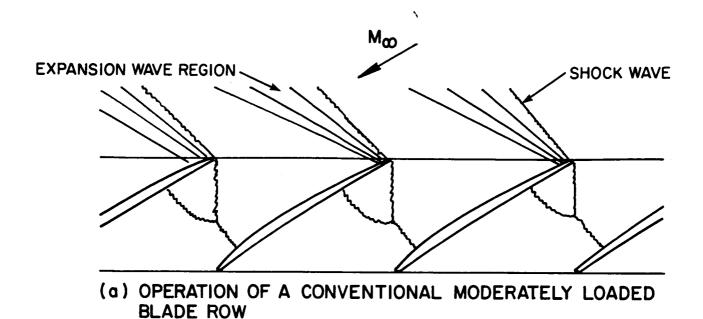


FIG. 7. CONICAL SHOCK FIELD FROM A ROTOR BLADE WITH A COMPOUND SWEEP LEADING EDGE.



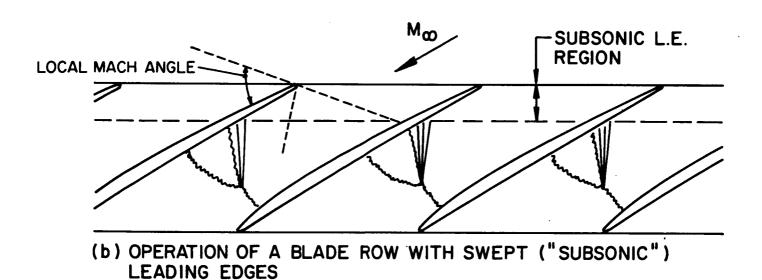


FIG. 8. COMPARISON OF THE OPERATION OF A MODERATELY LOADED BLADE ROW WITH AND WITHOUT SUBSONIC LEADING EDGES.

the blade are formed sufficiently far back that the disturbance is entirely contained in the blade row, even during off-design operation.

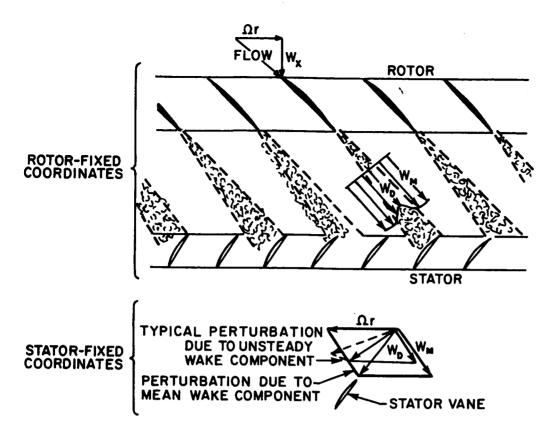
Using a swept leading edge also helps reduce the response of the rotor to inflow distortions, because the magnitude of the response is largely determined by the velocity component normal to the leading edge (Ref. 10). The effect of inflow distortion is most important near the tip of the rotor where the relative velocity is highest. Fortunately, the concept for sweeping the blades requires the most sweep near the tip.

3.2 Stator Noise Reduction by Leading Edge Sweeping and Blade/ Vane Number Selection

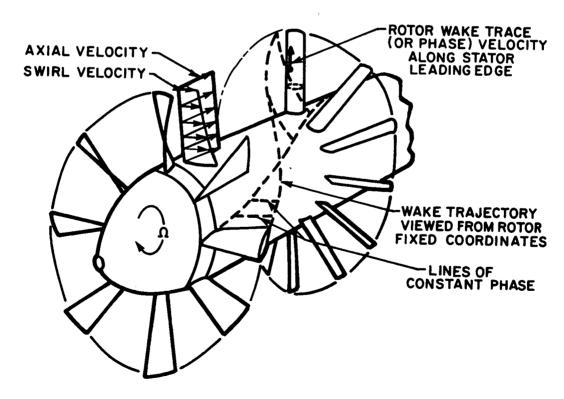
Although increasing the spacing between the rotor and stator leads to some noise reduction, the aerodynamic interaction between the rotor wakes and stator vanes remains an important noise source. Further reduction by conventional means can be achieved by choosing the proper number of blades and vanes to cut off many of the acoustic spinning modes in the duct (Ref.11). When the rotor tip speed is subsonic, the blade and vane numbers can be chosen so that all the spinning modes at blade passage frequency, and at least some of the modes at higher harmonics, are cut off. However, if the rotor tip speed is supersonic, at least one spinning mode at blade passage frequency cannot be cut off, regardless of the choice of blade and vane numbers. Since supersonic spinning speeds often occur on transonic fan designs, other means of stator noise reduction are of considerable interest.

Figure 9a illustrates the interaction of a row of stator vanes with rotor wakes when viewed on a surface of constant radius from the fan axis. The wakes can be described as flow regions with an average velocity \overline{W} lower than the velocity of the adjacent fluid, upon which a turbulent perturbation velocity field ΔW is superimposed.

Figure 9b shows a sketch of a three-dimensional wake/vane interaction in a fan. The structure of the viscous, usually turbulent, wakes that trail each rotor blade is complex. However, on the average, these wakes can be considered as being convected with the mean flow in which they are imbedded. The nature of the downstream mean flow is such that the convection process will distort the wakes from their original shape; namely, the downstream flow is distorted both axially and circumferentially across a given radial path, leaving the downstream pattern of the wake disturbance very much altered from the pattern at the rotor trailing edge. Suppose, for instance, the rotor is designed to give a mean flow that has a uniform axial velocity distribution and a free vortex tangential velocity distribution. Assuming



(a) The Interaction of the Stator Vane Row with the Mean and Unsteady Rotor Wake Components as Seen on a Constant Radius Surface.



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(b) A Sketch Showing the Three-Dimensional Nature of the Rotor-Wake/ Stator-Vane Interaction.

FIG. 9. THE CHARACTERISTICS OF ROTOR-WAKE/STATOR-VANE INTERACTION.

the wakes are radial at the rotor trailing edge, it is clear that the tangential velocity component will act to skew the wakes over, with the hub region leading the tip region. This situation is illustrated in Fig. 9b. In this case, the interaction of a given wake with a given stator vane does not occur simultaneously all along the stator vane span. Instead, the instantaneous spanwise interaction region of a single rotor wake will extend over only a portion of any one vane and will sweep along the vane leading edge, beginning at the hub and ending at the tip. The skewing of a wake due to convection by the downstream mean flow can be sufficient to involve simultaneously portions of several stator vanes.

The shape of wake and the magnitude of its velocity components vary from hub to tip. To complete this picture of the downstream flow field, one must consider the unsteady velocity components which account for the turbulent structure of the wakes and for any other sources of inhomogenieties in the flow, e.g., inlet flow distortions, large-scale flow instabilities, and blading errors. In general, the statistical properties of these unsteady components can be expected to vary axially, circumferentially, and radially.

Both the mean and unsteady velocity components of the wake flow induce unsteady loads on the stator vanes. The mean component will produce a load distribution that travels from hub to tip, changing shape and amplitude in accordance with the radial variation of the mean flow properties and wake strength, width, and skew. Imposed on this traveling load distribution will be the unsteady effect of the turbulent structure of the wake. The end result of all sources of unsteady loading on the stator vanes is to produce tonal and broadband noise. The tonal noise is usually considered to be the more important noise source. The speed at which the point of interaction of the flow disturbance with the vane travels along the span is called the trace speed.

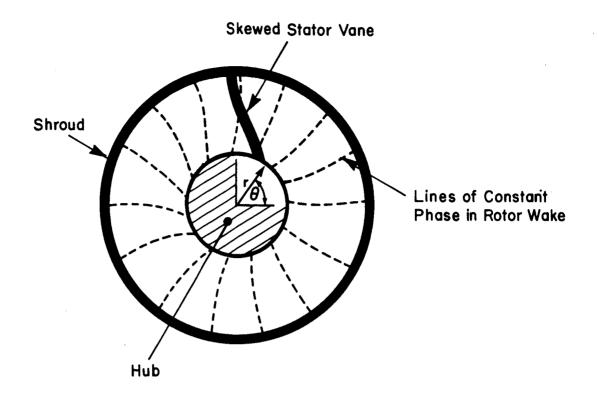
A particular source of unsteady loading will produce no significant acoustic radiation if it satisfies a subsonic and non-accelerating trace speed criterion along the vane span. The trace speed concept has been previously recognized for the problem of helicopter-blade/vortex interaction by Widnall (Ref. 12) although it has not been generally recognized in the study of fan noise.

The interaction of the wake with the vane produces a load distribution that travels along the vane. Suppose the vane is much longer than an acoustic wavelength. Following the trace of a phase front of this load distribution, acoustic radiation can occur along the vane span if the magnitude of the load changes,

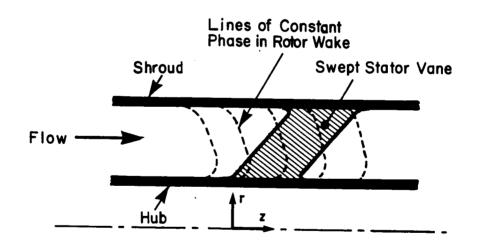
the phase speed changes with time, or if the phase speed is supersonic. For instance, in fan noise analyses the rotor-wake/ stator-vane interaction is usually assumed to be two-dimensional (corresponding to infinite spanwise trace speed). The conditions mentioned above are necessary for radiation but not sufficient. The interaction with the acoustic field produced by the other vanes must also be considered before the actual occurrence of acoustic radiation can be established. Therefore, regions along the stator vane span can be expected to be poor radiators if the phase speeds are subsonic, nearly constant, and local levels do not vary rapidly. Other regions may or may not be efficient radiators depending on the behavior of the distribution of sources elsewhere on the stator. Furthermore, end effects at the hub and tip (within approximately one half an acoustic wavelength of the ends) makes these regions potential radiators. These considerations are discussed in Appendix C, and justified in detail in Bliss, et al., (Ref. 13).

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Understanding the rotor-wake/stator-vane interaction and the criteria for radiation from the span of a single vane suggests ways in which the vane configuration can be altered to achieve noise reduction. The vane should be shaped so that loads traveling along the span move at a constant subsonic speed. Assuming that the amplitude of the load distribution. moving with a phase front, is essentially constant, then radiation from the vane span will not occur (except for end-The condition of a constant subsonic spanwise trace speed can be achieved by sweeping or skewing the stator vanes, as illustrated in Fig. 10. In this illustration, the lines of constant phase can be considered to be the intersection of the rotor wakes with the plane of observation (e.g., the r-0 plane in Fig. 10a, and the r-z plane in Fig. 10b). Except for the effect of shape changes, these lines travel at constant speed (rotational in the $r-\theta$ plane and rectilinear in the r-zplane) because of the rotation of the rotor. The speed at which a phase front traces the leading edge of the stator vane depends on the shape of the leading edge and the shape of the phase front. Clearly the trace speed can be controlled by either sweeping or skewing the stator vane. With this approach, radiation from the stator span can be prevented, leaving only acoustic radiation from end effects at the hub and tip of the Radiation from the hub region can be cut off by the proper choice of blade and vane numbers, provided that the rotation speed of wakes at the hub is subsonic. Since the rotation speed of wakes at the stator tip will usually be supersonic for a transonic fan, the radiation from tip end effects can never be entirely cut off. Note that the rotor



(a) Schematic Axial View $(r-\theta \text{ plane})$ -Skewed Vane



(b) Longitudinal Section (r-z Plane)-Swept Vane

FIG. 10. SCHEMATIC OF PHASE SHIFTS BETWEEN ROTOR WAKES AND SKEWED/SWEPT STATOR VANE.

wake pattern rotates with the same angular velocity Ω as the rotor. Thus, at any given radius at any downstream location between the rotor and stator, the rotation speed of the wake pattern is simply, Ω r, which is different than the swirl velocity component. This can be best visualized from rotor fixed coordinates from which the wake pattern appears "frozen."

Another, but related, way to view the effect of sweeping or skewing the stator vanes is as follows. Tyler and Sofrin (1962) have shown that for a given circumferential mode number, m, and hub-to-tip ratio, v, the radial structure of an acoustic spinning mode can be described by an infinite series of characteristic functions. The functions in this series differ according to their radial order, µ, i.e., each function has a different number of nodes in the interval between the hub and tip. The spinning speed at which each of these functions begins to radiate is always supersonic and increases with increasing radial order. Therefore, at a given supersonic spinning speed and fixed m and σ , only a certain number of the functions corresponding to the lowest radial order will not be cut off. Vanes can be skewed or swept so that the number of wakes on a given vane is increased, raising the radial order of the load distribution on the vanes. acoustic energy is thereby redistributed to higher radial orders, some of which will be cut off. The relationship between duct mode cut off and the constant subsonic trace criterion is discussed by Bliss, et al., (Ref. 13), and in Appendix C.

SECTION 4

FAN STAGE DESIGN SUMMARY

An experimental transonic fan stage was designed and constructed using the noise reduction concepts explained in the two preceding sections. The fan uses compound sweep rotor blades designed to have "subsonic leading edges" in the region of supersonic relative inflow. The stator vanes were swept back to achieve a constant subsonic trace speed of rotor wakes along the vane span. Figures 11a, b and c show photographs of the actual fan stage. A cross-sectional view of the fan as it will appear when installed in the test facility of NASA Lewis is shown in Fig. 12. As indicated in the illustration, the fan will be tested in both forward and reverse installation arrangements in order to measure both the fore and aft noise characteristics. The design data for the fan stage is summarized in Table 2. In the remainder of the report, the detailed design procedures used in the development of the fan stage are described.

TABLE 2. FAN STAGE DESIGN SUMMARY

Stage Characteristics:

Stage Pressure Ratio, $P_4/P_1 = 1.6$ Mass Flow Rate, W = 31.2 kg/s (68.8 lb/sec)Specific Mass Flow Rate:(referred to annular area at rotor inlet) $W_{as} = 199.03 \text{ kg/s} \cdot \text{m}^2 (40.76 \text{ lb/sec} \cdot \text{ft}^2)$

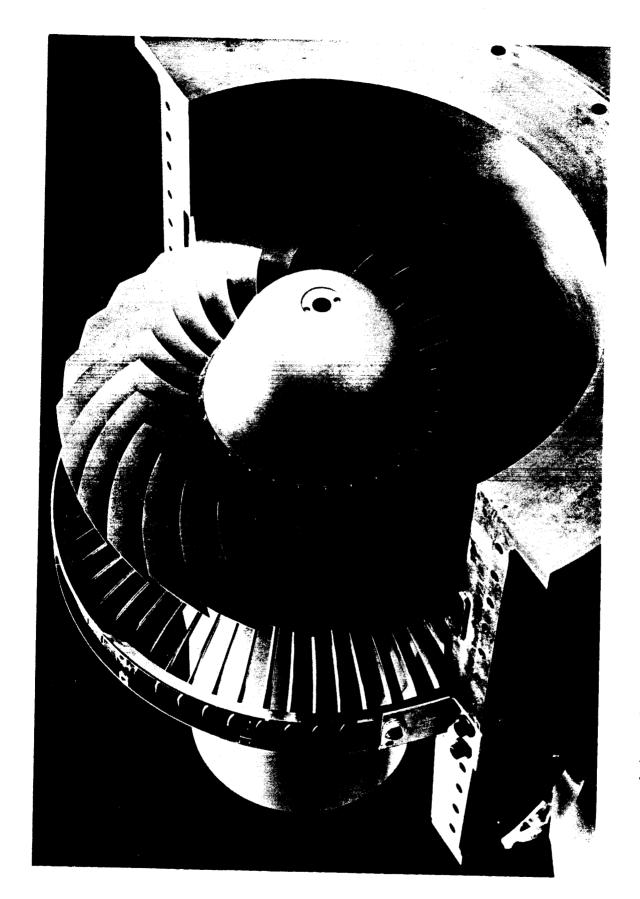
Polytropic Stage Efficiency, $\eta = 0.86$

Rotor:

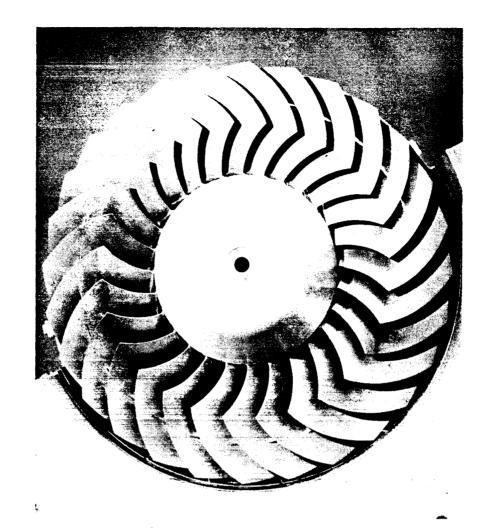
28 Compound Sweep Blades
Leading Edge Normal Mach Number = 0.91
Tip Speed = 480 m/s (1575 ft/sec)
Relative Tip Inlet Mach Number = 1.593
Rotor Inlet Tip Radius = 249 mm (9.803 in)
Rotor Inlet Hub-Tip Ratio = 0.442
Rotor Pressure Ratio, P₂/P₁ =1.64

Stator:

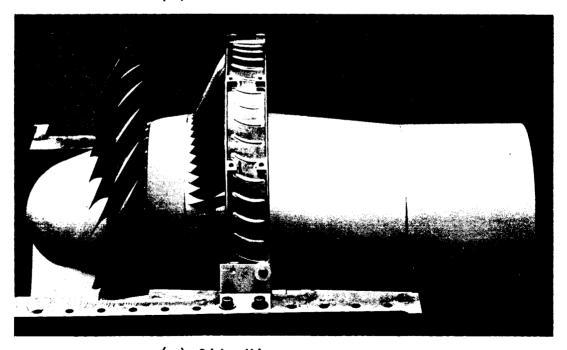
59 Swept Back Vanes Sweep Angle = 25° At Root, 40° At Tip Stator Inlet Mach Number = 0.80 Stator Pressure Loss $\Delta P_{3-4}/P_3 = .025$



(a) 45° VIEW OF ROTOR AND STATOR ASSEMBLY FROM FRONT OF FAN FIG. 11. PHOTOGRAPHS OF THE EXPERIMENTAL FAN STAGE

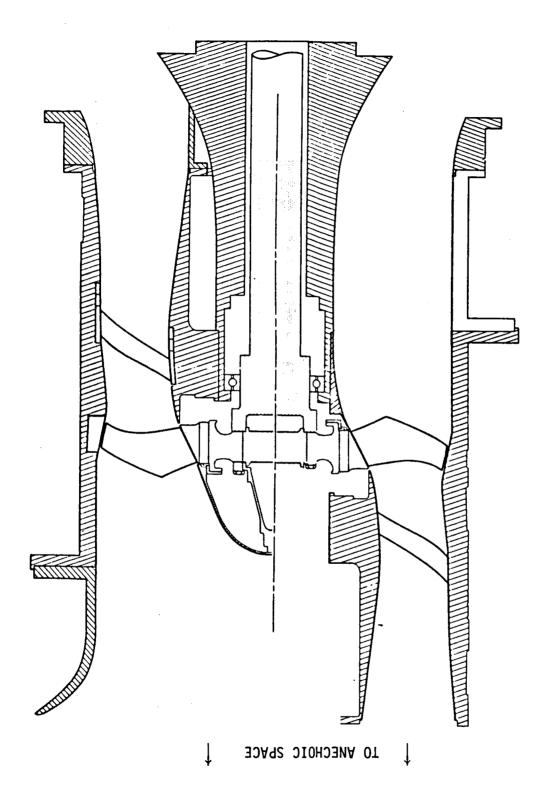


(b) Front View



(c) Side View

FIG. 11 concluded PHOTOGRAPHS OF THE EXPERIMENTAL FAN STAGE.



FORWARD AND REVERSE INSTALLATION IN THE W2 FAN NOISE TEST FACILITY AT NASA LEWIS.

SECTION 5

DETAILED ROTOR DESIGN

This section deals with all aspects of the detailed analysis and design procedures associated with the fan rotor. The aerodynamic design of a transonic rotor having blades with "subsonic" leading edges differs significantly from conventional design practice because the acoustic, aerodynamic, and structural requirements interact strongly with each other from the very beginning of the preliminary design phase. Therefore it was necessary to conduct numerous aerodynamic-acoustic-structural design iterations to optimize and finalize a rotor configuration satisfying all the design requirements.

The overall design point data for the rotor are listed in Table 2 of the previous section.

5.1 Aerodynamic Design

Certain differences are to be expected in the aerodynamic behavior of a rotor with subsonic leading edges. Since the entry into the blade row is shock free, any shocks that occur must remain within the blade row under all operating conditions because the edge region cannot support a shock system. Furthermore, the effects of sweep may introduce other three-dimensional flow phenomena which are not present in a conventional blade design. Given these facts, the rotor aerodynamic design was undertaken using the best conventional design practice combined with an anticipation of the most important effects of swept edges. The design was carried out primarily with the use of an axisymmetric flow computer program. Conventional (twodimensional) methods for analyzing the flow behavior within the blade row are not really adequate for the three-dimensional case of blades with swept "subsonic" edges. To handle this problem analytically requires a more general approach. Some work was done to adapt a new fully three-dimensional computer code to the analysis of flow through the blades with swept edges, but was discontinued due to schedule requirements.

An important question in the design of a rotor with "subsonic" edges is related to its surge margin. Because the edge region cannot support a shock system it was felt that the surge margin might be reduced. Such a reduction would occur, if the effective rotor operating range were limited by the condition that the shock system remain within the covered cascade region. The flow configuration in which the shock system must remain within the covered cascade, however, does not yield the maximum static pressure rise achievable in a conventional transonic-supersonic rotor. Consequently, in a

stage where surge is not triggered prematurely by the stator flow conditions, a rotor with subsonic leading edges might result in a decrease of the surge margin as compared to a conventional design.

Since the rotor aerodynamic loading essentially depends upon the rotor static pressure rise, the selection of the meridional flow path was the main design step taken to achieve the desired loading levels. Meridional channel conicity and curvature through the rotor section were traded off in several preliminary design attempts. The flow calculations were performed by means of a code which solves the general equation of radial equilibrium on straight axial or slanted stations for the axisymmetric flow case taking into account the radial variation of the blade efficiency. The polytropic efficiency η assumed for the rotor blading is shown in Fig. 13, where η is derived from

$$\frac{P_{2}}{P_{1}} = \left(\frac{T_{2}}{T_{1}}\right)^{(\eta)\left(\frac{\gamma}{\gamma-1}\right)}$$

It was found that the comparatively large channel conicity across the rotor section required by the high design pressure ratio $P_4/P_1=1.6$ and the wall curvature needed at rotor exit to prevent excessive channel contraction in the free space between rotor and stator, combine to shift the maximum rotor static pressure rise from the tip towards the midspan location, where the shock system has the greatest tendency to move upstream into the uncovered cascade region because of the lower relative inlet Mach number. The main preliminary design effort consequently was directed toward minimizing the static pressure rise at the critical midspan location.

The optimum channel configuration is shown in Fig. 14. The flow conditions are summarized on the Aero design program (R-121) input and output printout attached in Appendix A.

Figure 15a shows the distribution of the rotor static pressure ratio P_2/P_1 over the channel height, together with the relative inlet Mach number $M_{\rm w}$ and the corresponding normal shock pressure ratio \hat{P}_1/P_1 , which is roughly equivalent to the static pressure ratio obtained in the front portion of a conventional cascade with a normal shock attached to the leading edge.

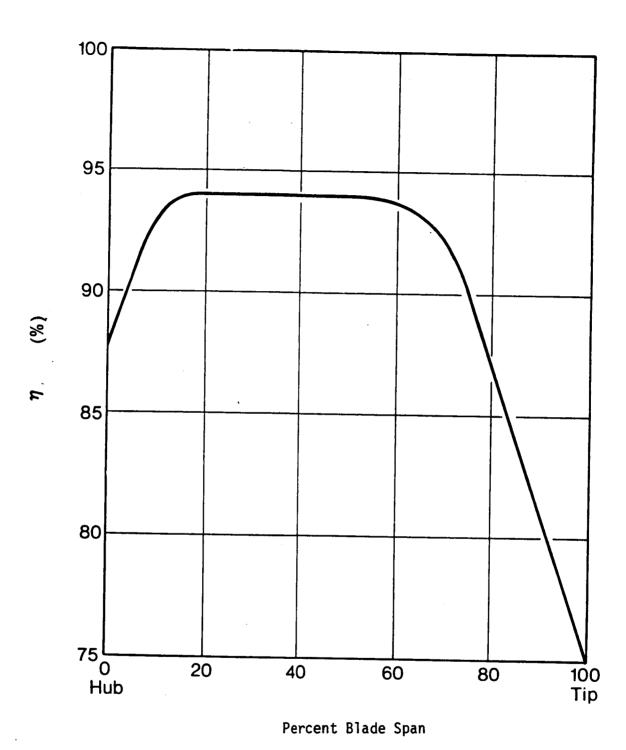


FIG. 13. ROTOR POLYTROPIC EFFICIENCY η

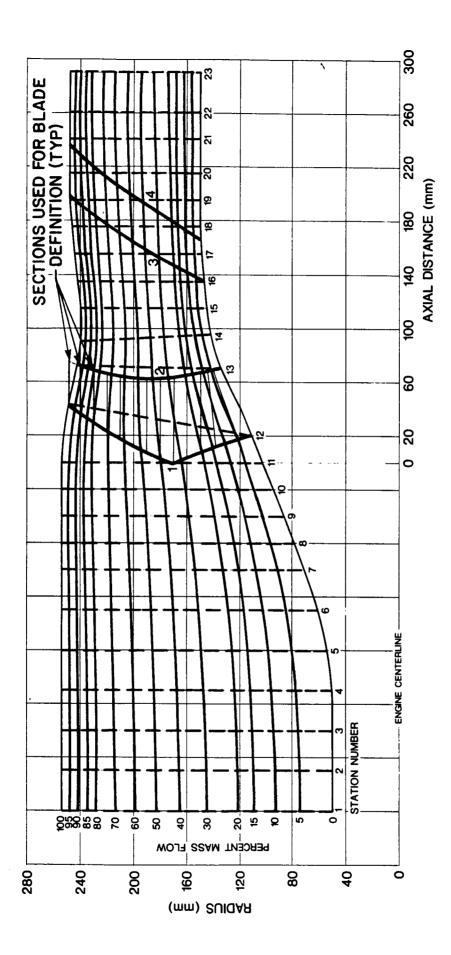


FIG. 14. MERIDIONAL FLOW PATH.

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Three types of operating conditions can be distinguished along the rotor blade span.

- (a) From the hub to the sonic radius $r_1(M_{\widetilde{W}_1}=1)$, the rotor static pressure rise is achieved essentially by subsonic relative flow deceleration and centrifugation.
 - (b) From $r_1(M_{\widetilde{W}_1} = 1)$, to $r_1(P_2/P_1 = \hat{P}_1/P_1)$ the rotor static

pressure rise must be achieved through a channel-contained normal shock or a pseudoshock system followed by subsonic relative flow deceleration. The radial distribution of the rotor static pressure ratio P2/P1 determined how far this operating condition extends beyond the sonic radius. In the present case, it extends roughly from the 12% to the 40% mass flow streamline, or from 20% to 53% of the span, i.e., slightly beyond the point of sweep reversal. The maximum inlet Mach number in this blade section remains below the 1.3 level at which the interaction of normal shock with a turbulent boundary layer produces extensive flow separation ($\hat{P}/P = 1.8$). If minor flow separation does occur in the upper portion of this region, the flow will reattach to the blade because of the large solidity provided in the vicinity of the point of sweep reversal. Consequently, it is expected that the design flow conditions will be obtained over this critical span section by a shock configuration located in the forward, yet still covered portion, of the cascade.

(c) In the upper blade section, P_2/P_1 is smaller than \hat{P}_1/P_1 , and the shock system consequently moves progressively toward the rear portion of the cascade. Since no shock is attached to the leading edge, the flow conditions are essentially similar to those in the diverging section of a converging-diverging nozzle in the supersonic off-design operating range.

Figure 15b schematically shows the meridional projection of the rotor blade and the anticipated shock/pseudoshock interception area on its pressure and suction sides. The main question pertains to the rotor surge margin, i.e. the extent to which the tip region will be allowed to increase its pressure ratio beyond the design value by forward shifting of the shock configuration before (i) flow separation occurs at the hub, or (ii) the shock system at midspan is forced into the uncovered cascade region.

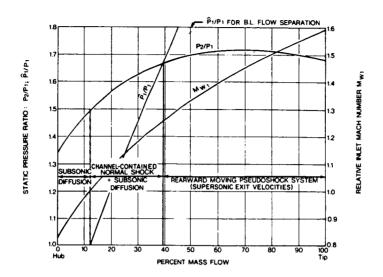


FIG. 15(a). ROTOR FLOW CONDITIONS: SPANWISE DISTRIBUTION OF STATIC PRESSURE RATIO AND INLET RELATIVE MACH NUMBER (Design Point).

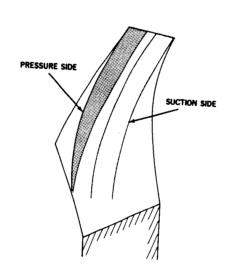


FIG. 15(b). ROTOR BLADE SHOCK/PSEUDOSHOCK INTERCEPTION AREA (Schematic).

5.2 Description of the Aero-Structural Design Interaction Problem

The problem of achieving acceptable stress levels is much more difficult for a rotor with swept leading edges than for a conventional design. The aerodynamic and structural requirements for the rotor blade are therefore closely coupled. the aerodynamic constraints, a number of design iterations were required to achieve acceptable stress levels and to optimize the The major aerodynamic constraints are that the rotor meet the design performance requirements and that the normal component of flow to the leading edge be maintained at a certain subsonic value. Because of the gradient of relative inflow Mach number, the angle of sweep must increase toward the tip to meet the condition of a subsonic normal component. In the present case, the maximum normal component Mach number was chosen to be 0.91 along these leading edges (actually lower near the hub). The value of 0.91 was chosen as a goal since it represented a normal Mach number sufficiently below sonic to avoid thicknessrelated shocks. Lower values can be chosen, but the severity of the blade leading edge excursions increase as the normal Mach number is lowered. For the fan design tip speed, the excursions of the swept leading edge are large and it was necessary to use a compound sweep configuration to minimize bending stresses. major variables available to control blade stresses are the location of the sweep reversal point, the local section properties of chord length, maximum thickness, thickness distribution, and the stacking of the blade sections.

Because of the large leading edge excursions, the centers of gravity of the blade sections can no longer be stacked on a radial line. In addition to the centrifugal tensile stresses, large bending moments about both principal axes of inertia of the blade sections were found to occur (Fig. 16). Achievement of acceptable stress levels required the use of a carefully chosen sweep reversal point and the development of an effective nonradial stacking procedure.

Typically, the most critical problem was the bending moments about the minor axis of inertia, and a special stacking procedure was used to minimize these moments. A near-optimum procedure for nonradial stacking is as follows. The blade sections were stacked starting at the tip and moving inward. The addition of each incremental blade section was made so that the center of gravity of the entire portion of the blade above this section falls on the axis of minimum inertia of the new section. The center of gravity of the new upper portion was then reevaluated before the next incremental section was added in the same manner. This procedure nearly minimizes the critical bending stresses around the axis of minimum inertia. The result is not completely optimum because of the

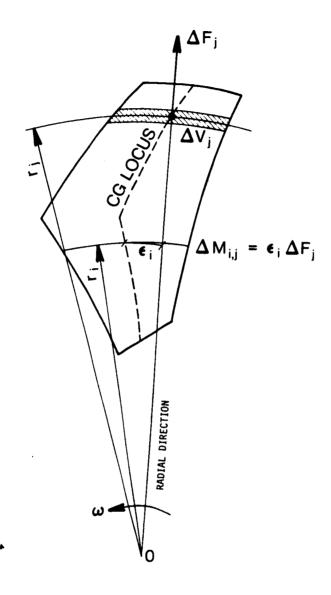


FIG. 16. BLADE CENTRIFUGAL FORCES AND MOMENTS.

complexity of the actual situation in which the stresses are determined by the complex interaction of many effects. Further improvements were made by iterative changes around the result of the above stacking procedure, particularly with the intention of relieving local stress concentrations. To the extent that high stresses arise due to bending around the axis of maximum inertia, these can be relieved largely by changing the location of the sweep reversal point and varying the local section chord and thickness.

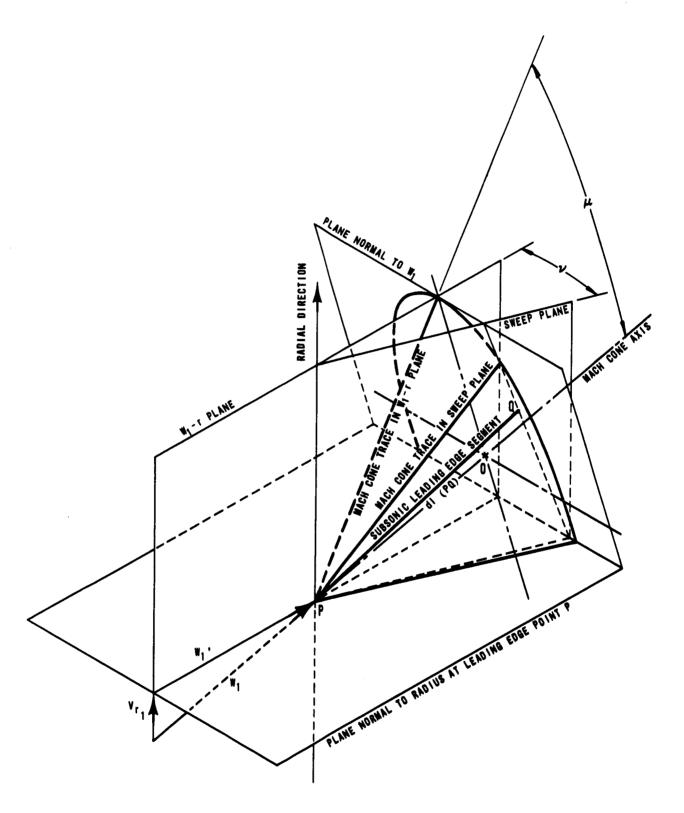
5.3 Determination of the Subsonic Rotor Leading Edge Geometry

At each leading edge point, the relative Mach vector $\mathbf{M_{w}}_{1}$ defines a Mach cone. To a prescribed value of the subsonic velocity component $\mathbf{M_{w}}_{1}$ perpendicular to the leading edge,

there corresponds a coaxial cone with smaller aperture. The subsonic leading edge elements must only satisfy the condition that they lie on such cones. Their direction otherwise is arbitrary.

Referring to Fig. 17, a particular sweep direction can be defined by specifying that each leading edge element be swept in the plane formed by the relative inlet velocity, W, and the radius (W-r) plane. This yields the shortest leading edge line from hub to tip, since it maximizes the radial projection of every leading edge segment.

Sweeping in the W-r plane however, does not result in a blade with minimum stresses. The resulting stacking of the CG's of the profiles in fact was shown to generate substantial bending moments around their axis of minimum inertia. The main parameter used to minimize bending stresses is the lateral sweep angle, ν , between the radial plane passing through the leading edge element dl and the W-r plane. The situation is illustrated in Fig. 17. The geometric analysis used for this design is described in Appendix B, and only some pertinent results are cited below. They are expressed by the two following equations for the cylindrical coordinates θ_L and z_L of the leading edge points in function of the relative flow angle β_1 , the lateral sweep angle ν , the slope ϵ_W of the relative velocity and the projection $\mu^{\text{\tiny H}}$ of the Mach cone angle μ in the W-r plane (see Appendix B):



E

u

V

FIG. 17. SONIC SWEPT LEADING EDGE ELEMENT.

$$\theta_{L}(r) = \theta_{L_{1}} + \int_{r_{M_{w}}}^{r} \left[\frac{\cos(\beta_{1} + \nu)}{\tan(\mu'' + \epsilon_{W_{1}})} \cdot \frac{1}{\rho \cos \nu} \right] d\rho \qquad (1)$$

$$Z_{L}(r) = Z_{L_{1}} \pm \int_{r_{M_{w}}}^{r} \left[\frac{\sin(\beta_{1} + \nu)}{\tan(\mu'' \pm \epsilon_{W_{1}})\cos \nu} \right] d\rho$$
 (2)

$$\varepsilon_{W_1} = \sin^{-1}\left(\frac{V_{r_1}}{W_1}\right) \tag{3}$$

The relation between μ " and μ is given by the formula

$$\tan \mu'' = \frac{\pm \sin \epsilon_{W_1} \cos \epsilon_{W_1} \tan^2 \nu + \sqrt{\tan^2 \mu (1 + \sin^2 \epsilon_{W_1} \tan^2 \nu) - \cos^2 \epsilon_{W_1} \tan^2 \nu}}{1 + \sin^2 \epsilon_{W_1} \tan^2 \nu}$$

$$(4)$$

In the above relations the (+) sign applies for backward, the (-) sign for forward sweep.

The formulae define a sonic leading edge, i.e., leading edge points lying on the Mach cones of the adjacent points. A subsonic leading edge is simply obtained by using in the formulae μ values corresponding to relative Mach numbers increased by a factor $f = 1/M_{1L}$, i.e. $M^{1}_{W_{1}} = M_{W_{1}/M_{W_{1}}}$ where

M is the subsonic Mach number of the relative velocity L component perpendicular to the leading edge. This simple relationship is illustrated in Fig. 18.

The second design parameter used to minimize the bending stresses was the sweep reversal radius. By proper selection of the point of sweep reversal, the center of gravity of the blade can be located in such as way as to project radially on, or near, the axis of maximum inertia of the hub section. From a structural viewpoint, the compound sweep blade of Fig. 6 could be considered as a blade with hub and tip sections designed and stacked according to conventional practice and fitted with an additional front section to materialize the subsonic leading edge configuration. The above CG stacking condition then could be fulfilled by similarly fitting a rear section to restore the symmetry of the mass distribution with respect to the axis of maximum inertia of the profiles. This, however, would maximize the additional blade mass and the elongation of the profile chord lengths required by the compound sweep design, which is structurally and aerodynamically undesirable. Proper selection of the radius of sweep reversal thus is necessary to ensure minimum blade stress and aerodynamic performance penalties. Adjustments of the profile chord lengths can be used only to compensate for a slightly off-optimum location of the sweep reversal point. Accordingly, the optimum stacking should yield hub stresses exceeding those of a conventional blade only by the contribution due to the blade mass added to incorporate the subsonic leading edge configuration.

From the preliminary design iterations, the meridional projection of the subsonic leading edge line and its sweep reversal point were known with sufficient accuracy to define the radial distribution of the relative Mach numbers M_{ν_1} (r) and the relative flow angles β_1 (r) and ϵ_{ν_1} (r) at the leading edge for final design. Those data were interpolated on the streamlines between stations 9, 10, 11, 12 of the R-121 flow calculation. (For the axial station nomenclature, refer to Fig. 14 and Appendix A.) Table 3 presents the interpolated inlet Mach numbers M_{ν_1} , together with the selected Mach factors f and the corresponding Mach numbers M_{ν_1} of the relative velocity component perpendicular to the leading edge and M_{ν_1} of the relative velocities, introduced in Eqs. 1, 2 and 4.

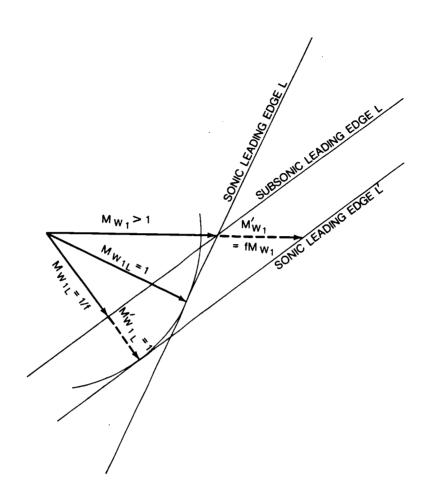


FIG. 18. SONIC AND SUBSONIC LEADING EDGES.

TABLE 3. Interpolated Aerodynamic Data for Final Subsonic Leading Edge Design

Leading Edge Radius (mm)	Relative Mach Nr. M _{wl}	Mach Factor f	$M_{w_1}^i = f M_{w_1}$	M _{wl} L
110	.829	1.206	1.000	.829
116	.859	1.170	1.005	.855
122	. 888	1.140	1.012	.877
129	. 924	1.118	1.033	.894
136	. 959	1.107	1.062	. 903
143	. 994	1.102	1.095	.908
150	1.028	1.100	1.131	. 909
160	1.078		1.186	
170	1.127	, .	1.240	
180	1.185		1.304	
190	1.242		1.366	
200	1.302		1.432	
210	1.363		1.499	
220	1.422		1.564	
230	1.481		1.629	
240	1.539		1.693	
249	1.588	1.100	1.747	.909

It will be seen that forward sweep starts immediately at the hub by setting $M_{W}' = 1$, i.e., by requiring that $M_{W} \equiv M_{W} = 1$

.829 at the hub section. The selected values of M $_{\rm W}^{\rm u}$ increase

gradually to .91 at approximately 1/3 of the span in accordance with the decreasing thickness and camber of the profiles, and then remain constant up to the tip section.

The cylindrical coordinates in the lateral sweep angle ν of the subsonic leading edge line are listed in the outlined columns of Table 4, which reproduces the input/output data of the computerized calculation.

5.4 Rotor Blade Profile Definition and Stacking Procedure

The optimum profile stacking configuration can be described as follows: At every blade section along the span, the CG of the upper blade portion projects radially on or near the axis of minimum inertia of that section. This means that the radial projections of the individual CG's of the upper profiles must straddle the axis of minimum inertia of the lower section (subsequently referred to as i-straddling). This is achieved by iterative selection of the lateral sweep angle ν along the span. During that iteration, the radial location of the point of sweep reversal initially selected is kept unchanged. When adequate i-straddling is obtained for all blade sections, the CG straddling with respect to the axis of maximum inertia (I-straddling) of the hub section is checked and the radial location of the point of sweep reversal modified accordingly.

The first preliminary design investigations were carried out with double circular arc profiles. In the course of the profile stacking iterations, it appeared that using airfoil sections with CG's shifting progressively backward in the lower span portion with forward leading edge sweep, and forward in the upper portion with backward sweep, i.e., a blade configuration with minimum chordwise excursion of the profile CG's, could substantially contribute to minimize bending stresses.

A simple analytical blade thickness distribution was used to simplify the design iterations involving changes in section properties to help relieve stresses. The thickness distribution is written in the following parametric form

$$t(x) = kx^{n} (c-x)$$
 (5)

CYLINDRICAL COORDINATES AND LATERAL SWEEP ANGLE OF THE SUBSONIC ROTOR LEADING EDGE LINE. TABLE 4.

AVEES INDICATOR	PANTIIS (4ETERS)	PELATIVE VELOCITY (M/SEC)	PADTAL VELOCITY (M/SFC)	RELATIVE . MACH NO.	ANGLE RETA(1) (DEG9.)	ANGLE NU (DEGR.)	ANGLE FPSILON(W (DFGR.)	ANGLE MU (DEGR.)	ANGLE MIJ (W-R.) (DEGR.)	FUNCTION THETA(R)	FUNCTION Z(R)	THETA	Z (WETERS)
7	0.11001	274.00	66.99	1.0900	36.80€	15.000	14.586	99.439	97.730	1.51448	0.21170	3.0	0.0
<u>-</u>	^.11699	טני שאל.	64.10	1.0151	36.000	15.200	13.024	84.283	84.261	1.90150	0.27434	-0.01037	-0.00148
7	0.1.200	CU-\$6.	25.56	1.0120	35.150	15.000	11.676	81.168	81.129	2.03814	3.29791	-0.02220	-0.00320
. 7	1,0621.0	375.57	63.53	1.1339	34.100	14.333	10.086	15.479	75.395	2.44191	0.35480	-0.03776	-0.00547
7:	0.13691	11.5°	4 A. DO	1.0620	33.050	13.000	8.695	70.326	70.202	2.84295	0.40108	-0.C5629	-0.00813
1	1430)	327.77	42.00	1.7957	32.150	62.70	7.379	65.957	65.855	3.24653	0.41436	-0.07758	-0.01100
7	2.15000	347,09	36.50	1.1310	31.600	5.390	6.163	62.150	62.114	3.65857	C-39872	-0.10177	-0.01386
7	0.14000	156.01	29.50	1.196	29.550	0.633	4.753	57.476	57.476	4.11398	0.38233	-0.14079	-0.01775
- •	N E E	8 F V	1 V > B 3			-2-119			53.740	4.40314	0.36670		
_	1.170.n	377.00	23.17	1.2400	28.130	9.300	3.535	53.751	53.596	3.06939	0.39463	-0.18346	-0-02150
_	1,1800)	301.57	18.50	1.3240	27.206	8.000	2.708	50.174	49.899	3,59490	0.44492	-0.15055	-3.01730
-	0.19nn	410°00	13.50	1.3660	26.590	7.000	1.987	47.060	46.885	3.87483	0.48729	-0.11363	-0.01263
	n.2000	429.57	7.50	1.4320	Jü6 *57	9.000	100.1	44.293	44.131	4.24864	0.52891	-0.07302	-0.00755
	נטיול".	449,57	3.51	1.4997	25.350	5.000	3.164	41.845	41.708	4.61 908	0.56784	-0.02868	-0.00206
`	00062.	447.50	-7.50	1.5640	24.650	4.200	-0.919	39.746	39.633	4.98041	0.60361	0.01932	0.00379
_	100k2*0	6484	-16.50	1.6290	J36°€∠	4.399	-1.946	37.870	37.752	5.33953	3.65024	0.07090	0.01005
_	rrn45.r	504,50	-27.50	1.6930	23.100	000**	-3.119	36.204	36.070	5. 73638	0.70451	0.12626	0.01682
	0.74907	621.69	-37.fc	1.7477	22.200	4.200	-4.069	34.918	34.770	6.08336	0.74535	0.17946	0.02335

I

where c is the chord length and n, a shape parameter. By adding a leading and trailing edge thickness,

$$t_{LE} \equiv t_{TE} = \tau \nu C$$

where τ is the LE and TE thickness factor and ν = t_{max}/C the relative blade thickness, a practical blade thickness distribution is obtained. The abcissa for maximum thickness is given by

$$x_{t_{max}} = \frac{nc}{n+1}$$
 (6)

the factor k by

$$k = \frac{v (1-\tau)}{\left(\frac{nc}{n+1}\right)^n \left(\frac{1}{n+1}\right)}$$

The complete non-dimensionalized formula is

$$\frac{t}{c} = \tau v + \frac{v(1-\tau)}{\left(\frac{n}{n+1}\right)^n \left(\frac{1}{n+1}\right)} \left(\frac{x}{c}\right)^n \left(1-\frac{x}{c}\right)$$
(7)

For n = 1, $\binom{x}{c}_t = .5$. Furthermore, the second derivative is constant, so that the resulting profile is essentially a double circular arc profile for small thickness.

For n > 1, $(x/c)_{t max}$ > 1/2 and the profile CG shifts toward the trailing edge. Since the first and second derivatives of the thickness distribution are continuous, the profile curvature is continuous.

Using profiles with circular mean camber lines and n varying from 1 to 1.8 from the hub to the point of sweep reversal, and back to 1 at the tip section, a favorable blade configuration was obtained. However, manufacturing difficulties and the extreme sensitivity to tolerance and foreign object damage of thin profiles with n > 1.5 lead to the selection of n=1, i.e., essentially double circular arc profiles for final rotor blade design.

The blade cascade geometry was defined by means of conventional procedures and criteria. Figure 19 shows representative streamline velocity triangles, together with the corresponding relative flow deceleration rates W_2/W_1 and static pressure ratios P_2/P_1 , the selected cascade solidities σ = c/s and the resulting D-factor values. The hub and tip cascade solidities are equivalent to those which would have been selected for a conventional design with identical rotor inlet and exit flow conditions. The 30% streamline velocity triangles are representative of the conditions at the sweep reversal section (r = 170mm).

The flow deviation angles δ at rotor exit were calculated with Carter's empirical formula (Ref. 13)

 $\delta = m\phi / \sqrt{\sigma} \tag{8}$

 Γ

r

U

M

with $m = 0.23 + 0.05 \beta_2$ (circular mean camber line). For small camber angles ϕ , Eq. (8) gives unacceptably low deviation angles, especially in transonic cascades with shock-boundary layer flow interaction. A minimum deviation angle of 2° was arbitrarily assumed and the calculated δ - values were faired gradually to the minimum value toward the tip section. actual profiles were defined on coaxial cylinders for the most part of the blade. Three profiles were defined on cones in the hub region to ensure a smooth evolution of the profile geometry toward the conical hub section. Fig. 20 shows the relative inlet and exit angles β_1,β_2 with the tangential direction and the deviation angles $^{1}\delta$ 2 used to define the cascade geometry. All profiles were set at a nominal incidence $i = +2^{\circ}$ with respect to the suction surface. The selected profile sections are indicated on Fig. 21. Table 5 lists the profile design data defining the cylindrical and conical sections unwrapped on planes tangent to the cylinders and cones. (While all angles are conserved in the development of cylindrical sections, the profile camber angle is reduced in the developed conical sections by the sector angle formed by the radii passing through the leading and trailing edge points.)

The coordinates of the center of gravity of a cylindrical section are determined by the following simple relations:

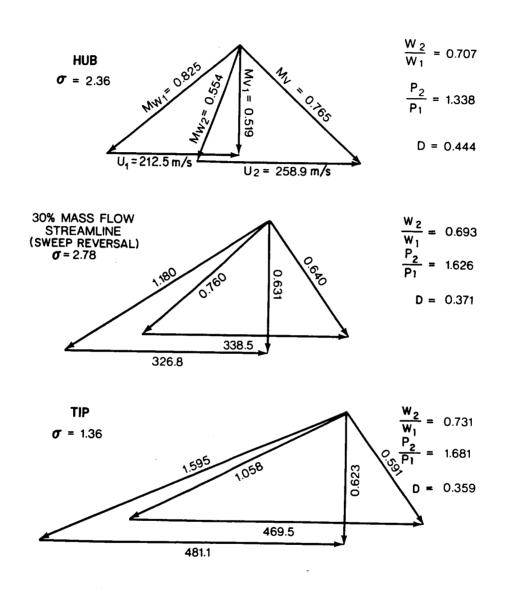


FIG. 19. ROTOR VELOCITY TRIANGLES (28 blades).

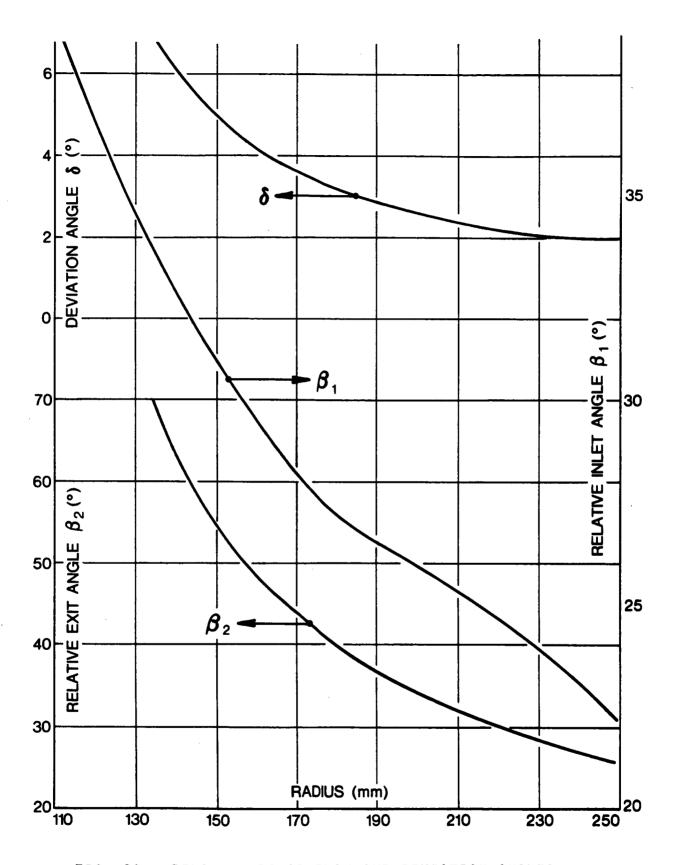


FIG. 20. RELATIVE ROTOR FLOW AND DEVIATION ANGLES.

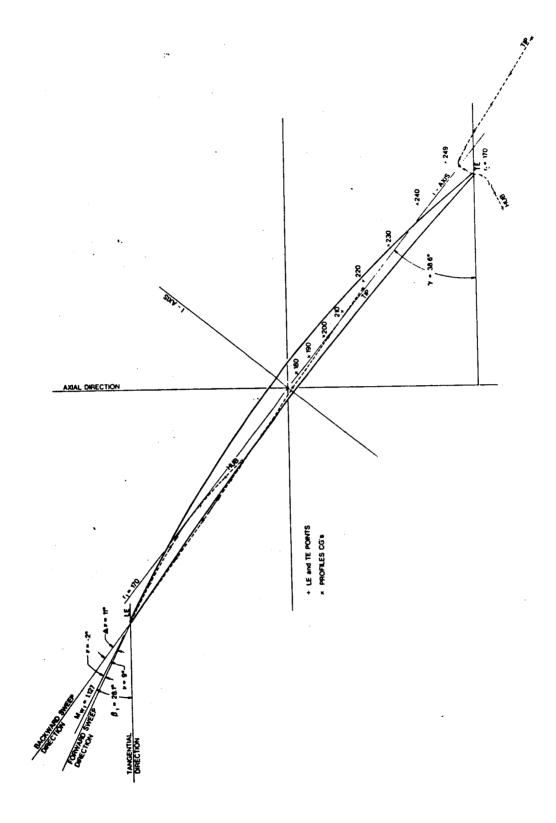


FIG 21. DEVELOPED CYLINDRICAL SWEEP REVERSAL SECTION WITH RADIAL PROJECTION OF PROFILE CG'S AND LE AND TE LINES.

TABLE 5. Rotor Blade Profile Data 28 Blades (Developed Cylindrical and Conical Sections).

F.

Section Radius (mm)	Mean Camber Angle ф (°)	Setting Angle	Chord Length c (mm)	Relative Thick- ness $\mathscr{N}(\%)$
110/134	28.85	62.44*	64.55	10.77
122/140	24.70	56.45*	67.91	9.73
136/145	24.95	50.93*	74.73	8.03
150	26.10	46.05	85.20	6.18
160	20.90	41.95	95.20	4.95
170	17.00	38.60	105.90	4.00
180	13.70	36.05	102.30	
190	10.90	33.95	97.70	ļ.
200	8.70	32.25	92.90	
210	6.90	30.75	88.20	
220	5.60	29.40	83.80	
230	4.70	28.25	80.20	
240	3.90	27.05	77.20	
249	3.30	25.85	75.00	4.00

^{*}Angle between chord and tangent to the developed section circle at the trailing edge

$$\theta_{\rm cg} = \theta_{\rm L} + \frac{.5c \cos \gamma + d \sin \gamma}{r} \tag{9}$$

and

$$Z_{cg} = Z_{L} + .5 c \sin \gamma - d \cos \gamma$$
 (10)

where c is the chord length, and d is the distance of the CG to the profile chord in the developed section.

Figure 22 shows the situation for a developed conical section. From the aerodynamic design, the geometric characteristics of the profile, especially the inlet and exit angles β_1 and β_2 between the tangent to the mean camber line at LE g and TE and the circumferential direction, are known. Also known are the inlet and exit radii r and r and the meridional projection c_m of the chord. Hence, from similar triangles in the meridional plane:

$$R_1 = \frac{c_m r_1}{r_2 - r_1}$$
 Further, $R_2 = R_1 + c_m$

and with $m = R_1 \sin \psi$ and $\delta R = R_1 (1 - \cos \psi)$

$$c^2 = (c_m + \delta R)^2 + m^2 = c_m^2 + 2R_1 R_2 (1 - \cos \psi)$$
 (11)

In the developed section, the camber angle is

$$\Phi = \beta_{2g} - \beta_{1g} - \Psi \tag{12}$$

and the setting angle is defined by

$$\sin \gamma = (c_m + \delta R)/c \tag{13}$$

Assuming a circular mean camber line in the developed section,

$$\beta_2 = \gamma + \Phi/2 \tag{14}$$

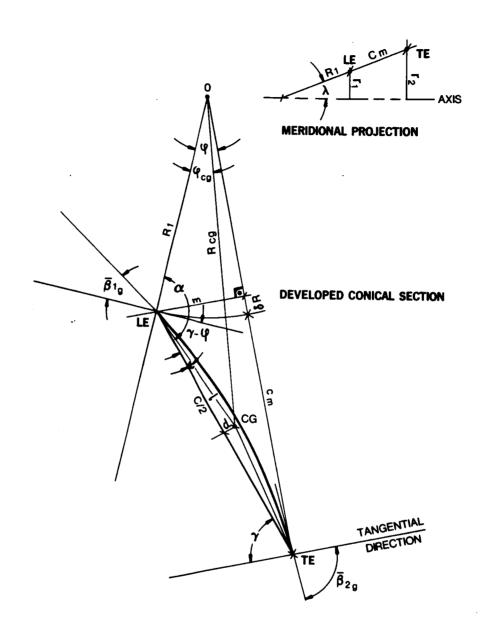


FIG. 22. DEFINITION OF CONICAL BLADE SECTIONS.

Equations (11)-(14) determine the four quantities c, ψ , Φ and γ . They must be solved by successive iterations. Assuming tentatively ψ , equation (11) gives c, equation (12) gives Φ , equation (13) gives γ , while ψ is iterated until equation (14) is satisfied.

After a profile is superimposed upon the circular mean camber line, CG distance d is known and the coordinates of the center of gravity are determined as follows:

$$\ell^2 = \frac{c^2}{4} + d^2$$
 (symmetrical profile), $\epsilon = \sin^{-1}(d/\ell)\alpha = 90 + \gamma - \psi - \epsilon$

Hence, $R_{cg}^2 = R_2^2 + \ell^2 - 2 R_2 \ell \cos \alpha$ and from triangle O-LE-CG:

$$\sin \psi_{cg} = l \sin \alpha / R_{cg}$$

Finally, $r_{cg} = \frac{r_1}{R_1} R_{cg}$ and the cylindrical coordinates of the center of gravity are

$$\theta cg = \theta_{L} + \frac{R_{cg} \psi_{cg}}{r_{cg}}$$
 (15)

$$Z_{cg} = Z_{L} + (R_{cg} - R_{1}) \cos \lambda$$
 (16)

All CG stacking investigations, including preliminary bending stress evaluations, were carried out manually. However, as will be discussed later, verification of stress levels was carried out using computer programs at BBN and AVCO Lycoming. Figure 23 shows the final stacking of the profile CG's radially projected on the conical hub section, which was investigated by NASTRAN analysis. The corresponding distribution of the lateral sweep angle v is shown on Table 4. The NASTRAN results indicated that the stress distribution at the hub section could be improved by a slight tangential shift of the first two conical sections in the rotation sense. $\Delta\theta_L$ - shifts of -.008 for the hub and -.004 for the next section were effected without readjusting the z - coordinates of the leading edge points. Those shifts are indicated on Fig. 23. Provision has been made in the i straddling to generate a moment that continuously compensates the moment of the aerodynamic forces, (which are reflected in results hereafter).

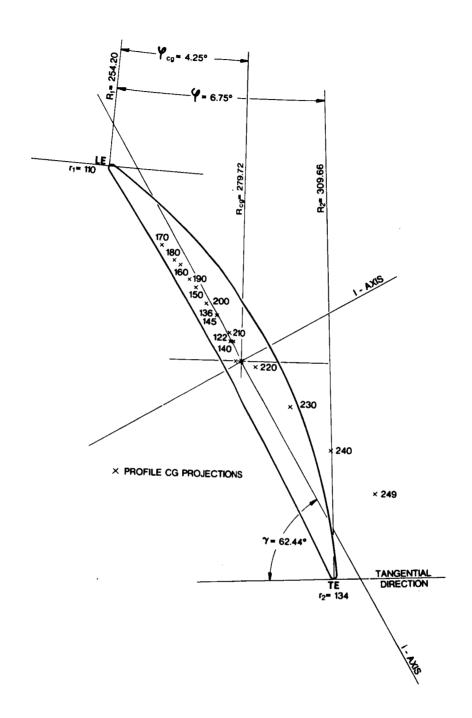


FIG. 23. CONICAL HUB SECTION DEVELOPED ONTO PLANE TANGENT TO CONE, WITH SUPERIMPOSED RADIAL PROJECTION OF PROFILE CG's.

The optimum radial distribution of the lateral sweep angle ν is different for forward and backward leading edge sweep directions. Consequently, a discontinuity of lateral sweep may occur at the sections above and below the point of sweep reversal. This, in turn, results in a high rate of curvature of the blade surface. Since the blade is defined by discrete sections, this appears only as a more or less pronounced concentration of the spanwise curvature of the blade surface in the sweep reversal region. Nevertheless, this local curvature increase generated prohibitive stresses near the trailing edge in several preliminarily generated configurations.

This problem was compounded by the additional bending moment around the I-axis of the section of sweep reversal, due to the rearward location of the CG of the upper blade portion with backward leading edge sweep. The difficulty increases since the sweep reversal was selected initially so as to minimize that moment and it was gradually moved inward from $r_{\rm sr}$ = 188 to 170 mm, still leaving the blade CG in a forward position with respect to the I-axis of the hub section. The stress concentration problem at the sweep reversal section was solved by means of an elaborate compromise of the profile stacking through that section, involving especially the selection of the critical lateral sweep angle discontinuity. For the final configuration, with r_{sr} = 170 mm, this was achieved at a late design state only, the last optimization step, which would have required the sweep reversal point to be set at 160 mm radius, or the profile chord lengths to be increased in the upper blade section. With the present stacking, the highest stress is 645 N/mm^2 (93.5 ksi), which is adequate for concept demonstration purposes. Figure 21 shows the developed sweep reversal section, together with the radial projections of the profile CG's of the upper blade portion and the leading and trailing edge lines. The upper profile CG's have been stacked to compensate for the aerodynamic moment and to minimize the additional TE tensile stress resulting from the rearward CG position of the upper blade portion.

Whereas the radial projection of the leading edge points indicates a smooth subsonic leading edge line, the trailing edge line does not appear to be as smooth as desirable. For manufacturing the blade was defined by flat sections generated from the blade configuration defined in the cylindrical

coordinates used for the stacking investigations. Any minor irregularities of the trailing edge were smoothed out by a slight increasing of the chord lengths of a few local sections. All profile data are listed in Table 5.

5.5 A Review of the Rotor Blade Design Iterations for Stress Optimization

The main objective of the preliminary design effort (see Fig. 24) was to define a stacking configuration that maintains the subsonic leading edge concept, i.e., satisfies the acoustic rotor design requirements with as low a blade stress level as possible. A target design goal of 725 N/mm² (105 ksi) maximum steady state stress was sought for the design speed of 18,450 rpm. For the selected titanium blade, such a stress level is considered adequate for the demonstration purposes of this program.

As a first step in each iteration, both manual and computerized beam-type stress computations were carried out to develop a feel for the iterative stacking procedure and to ensure numerical agreement. The standard AVCO Lycoming blade stress computer program which was used treats the blade as a twisted, rotating cantilevered beam with variable section properties, and takes into account the shroud and aerodynamic forces and the centrifugal restoring moments. All trial blade stacking iterations were analyzed with this program.

Simultaneously, a quick, inexpensive and efficient finite element analysis was used at BBN to verify the results of blade The program, based on SAP, was operated in coniterations. junction with a blade geometry generator which was based on the family of blade profile shapes, described previously by Eqs. 5-7, which reduce to a minimum the number of parameters required to specify a blade shape; namely, the leading and trailing edge coordinates, the section setting angle and camber, and the profile shape parameters. The program was therefore very well suited for iterative design studies. The purpose of the simultaneous effort was to provide further verification of the beam and manual analysis and to help identify stress concentration, which are neglected in the beam-type stress analysis program and in the manual calculations. These efforts were deemed necessary because the blade configuration differs radically from more conventional designs, and it was uncertain whether conventional design methods would be sufficiently accurate.

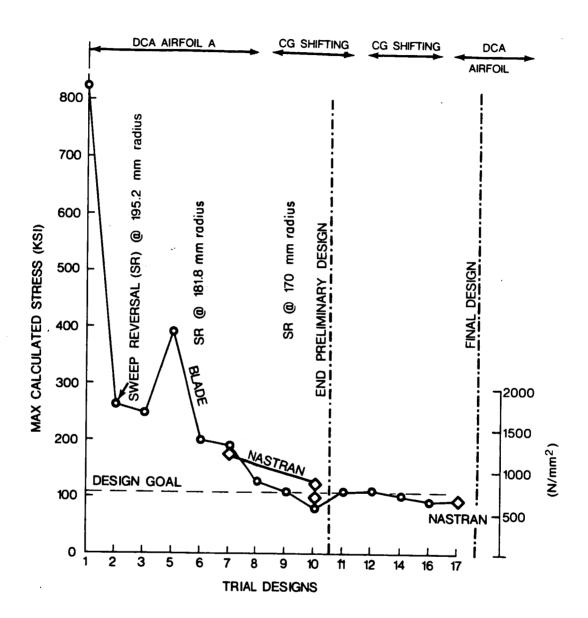


FIG. 24. OPTIMIZATION PROGRESS.

A NASTRAN stress analysis program was used by AVCO Lycoming on design iterations which were considered particularly important, and for the final stress computations verification.

The evolution of the maximum blade stress levels as the blade design evolved through the series of trial designs is shown in Fig. 24. The results of the first design substantiated the impractical stress level of a blade with simple forward leading edge sweep. The initial sweep reversal radius (SR) was selected at 195.2 mm. The stacking for trial design 2 was such that the center of gravity of each of the 13 cylindrical blade sections used to define the blade projected radially down onto the axis of minimum inertia (i-axis) of the airfoil section immediately below. For Iteration 3, all section CG's were projected onto the i-axis of the hub section. For Iteration 5, all section CG's above the sweep reversal section were projected onto the SR section i-axis, while the stacking of Iteration 3 was kept for the lower blade sections. As can be seen, the resulting misalignment of the upper blade portion with respect to the hub section produced higher hub stresses. However, this design also showed the lowest stress level for the upper blade portion.

For Iteration 6, the sweep reversal radius was lowered and the misalignment was corrected by introducing a discontinuity of the lateral sweep angle, (i.e., the angle between the sweep direction and the radial plane containing the relative inlet velocity), at the point of sweep reversal. By varying this parameter, a number of stacking combinations involving individual compromises within the upper and lower blade sections, were investigated. Iteration 7 shows the best result obtained with this stacking concept.

With the stress level still substantially beyond the preliminary design goal of 105 ksi, a detailed investigation of the stress pattern in design 7 was performed using the NASTRAN stress program. The excellent correlation which was obtained substantiated the beam-theory analysis method as a useful approach to analyze blade stacking changes.

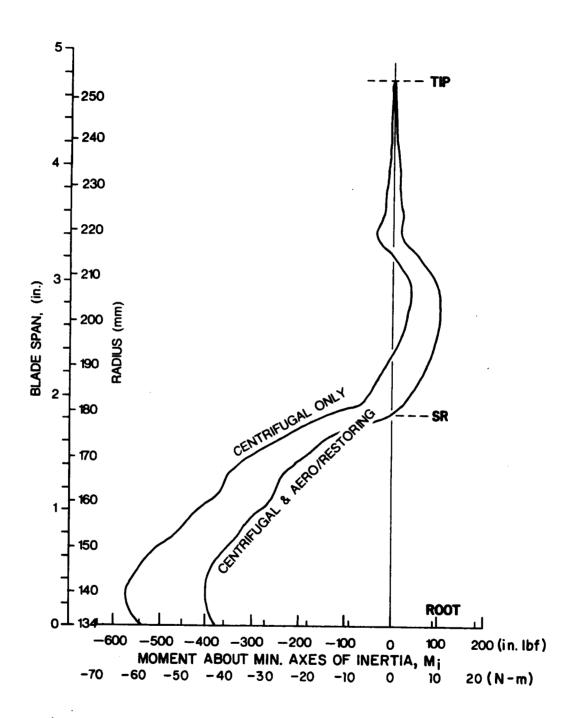
Subsequent iterations were conducted with the optimum stacking concept described in Sec. 5.4. This stacking satisfies the condition that, at every section along the span, the CG of the entire blade portion above the section projects radially onto the i-axis of the section. As shown by Iteration 8, this reduced the maximum stress level very nearly to the preliminary design target value.

The new stacking concept confirmed the necessity of a lateral sweep angle discontinuity at the point of sweep reversal to achieve proper stacking of the profile CG's across that section. This discontinuity, however, resulted in a rapid change of the spanwise curvature of the blade surface in the trailing edge region, which in turn results in a local stress concentration that was not shown by the simplified analysis. Iterations 1-8 were conducted with double circular arc profiles (DCA). Iterations 9 and 10 used new profiles featuring rearward CG shifts from the hub to the section of sweep reversal, and forward CG shifts from that section to the tip (see previous section). In this way, the CG excursions from a radial line were minimized within the leading and trailing edge envelope and the stresses were reduced to the target level.

Figures 25 and 26 show the moments about the axes of minimum inertia and maximum stress distributions for Design 10 as calculated by the standard blade stress program. The influence of aerodynamic loads and centrifugal restoring moments are also shown. (Design 10 was chosen for further study since this is the design which first indicated stresses below the design goal.)

A detailed investigation of Design 10 was also performed with the NASTRAN program. The results showed local high stresses of 96 ksi at the trailing edge of the sweep reversal section and 110 ksi at the leading edge of the hub section. By slightly increasing the chord length of the sweep reversal section, and slight re-alignment of the conical hub, these stresses were brought down to 84 and 96 ksi, respectively. The NASTRAN finite element representation of this configuration, called Design 10A, is shown in Fig. 27. The stress distributions of the suction and pressure surfaces are shown in Fig. 28.

During the entire iteration process, it was apparent that the radial location of the point of sweep reversal would have to be moved substantially inward from its initially assumed location in order to avoid a large moment about the I-axis of the hub section. Moving the point of sweep reversal inboard, however, increases the bending moment about the I-axis of the sweep reversal section, thereby increasing the tensile stress at the trailing edge of that section. To minimize the local trailing edge stress concentration the radial location of the sweep reversal section was moved inboard cautiously. Even so, the blade CG remained ahead of the hub section I-axis, and resulted in an additional bending stress (on the order of 120 N/mm²) at the hub section leading edge.



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FIG. 25. SECTION MOMENT DISTRIBUTION [Preliminary Design 10].

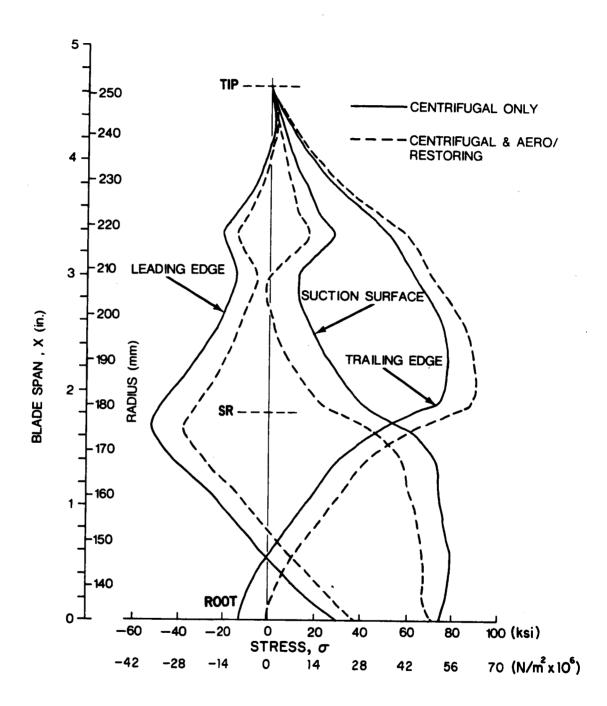


FIG. 26. MAX STRESS DISTRIBUTION [Preliminary Design 10].

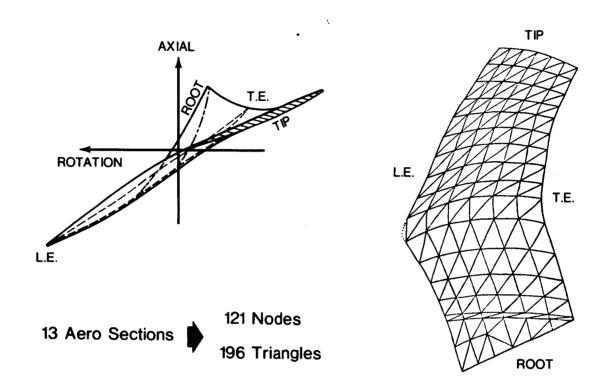


FIG. 27. NASTRAN ANALYSIS [Preliminary Design 10A].

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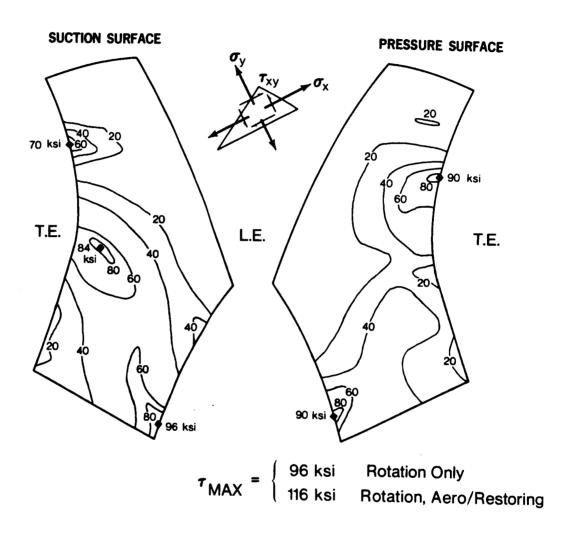


FIG. 28. EXAMPLE OF INTERIM RESULTS OF NASTRAN STRESS ANALYSIS MAX SHEAR CRITERION [Preliminary Design 10A].

Prior to the selection and analysis of the final blade design, several intermediate designs were investigated based on local shifts of the CG location within the indificual airfoil profiles. (Noted as Designs 11 through 16 in Fig. 24.) The polynomial blade sections had been evolving toward n=1 or a DCA profile. For manufacturing reasons, however, double circular arc profiles were specified for the final design. raised the stresses to virtually the level of iteration 9, and additional stacking iterations were required to achieve the design objective. In particular, the relative blade thickness was increased from 10.00 to 10.77% at the hub section. resulted in a 10% decrease in the stress level. Additional reductions were achieved through a judicious balancing of the profile stacking in the lower blade portion and lateral sweep angle discontinuity at the sweep reversal section.

A check was performed to see if the DCA profiles allowed adequate flow area margins. On an average basis, the rotor throat passage area has a large margin to sonic throat area because of the comparatively high mean relative inlet Mach number level \overline{M}_{W} , = 1.33 and the positive inlet incidence of 2° selected for optimum blading efficiency. The throat hub region is most susceptible to local throat choking because of the transonic inlet flow conditions and the higher relative blade thickness. Because of unknown 3-dimensional flow effects, it is difficult to determine local blade stream tube areas and no definite section throat area margins thus were specified for the design. A check, however, was tentatively made for the rotor hub section. On the two-dimensional basis of the developed section of Fig. 23 the ratio of throat to inlet passage width is At the throat location, however, the channel height has decreased from 139 to 136.3 mm. Assuming that all individual stream tube heights are reduced in the same proportion, the effective geometric throat/inlet area ratio thus is $A_{min}/A_{in} =$ $1.045 \times 136.3/139 = 1.027$. With a relative inlet Mach number of .825, the sonic area ratio A_{in}/A_{s} is 1.0285, thus A_{min}/A_{s} = 1.027 \times 1.0285 = 1.055, i.e., a 5.5% choke area margin.

In the hub region, the flow has the tendency to be deflected inwards because of the forward leading edge sweep. On the other hand, the increasing density toward the tip at rotor exit combines with the essentially constant axial velocity of free-vortex flow to shift the streamline pattern outwards at rotor exit. Those compensating effects cannot be quantified at the throat location and the comparatively large 5.5% margin thus was judged adequate to account for the possibility of unfavorable three-dimensional effects and for the suction side boundary layer

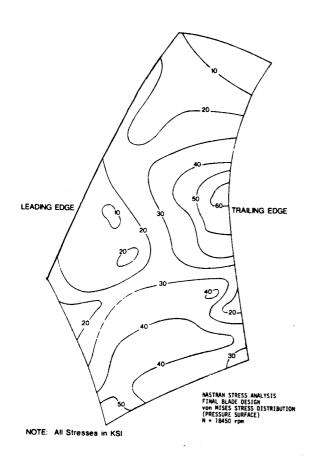
growth upstream of the throat in the absence of a detached leading edge shock. In summary, in spite of the selection of DCA profiles, the individual rotor section throat margins are adequate.

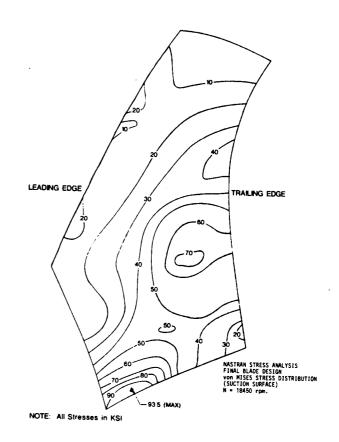
5.6 Final Rotor Blade Stress Analysis

The stress analysis for the final design iteration was performed using NASTRAN. The loads considered in this run were based on the maximum operating speed of 18,450 rpm. In addition to the major contribution of the centrifugal load, aerodynamic gas pressure loads, the centrifugal load and the torsional restraints of the part span shroud were applied to the blade. The resulting von Mises effective stress patterns over the pressure and suction surfaces of the blade are shown in Figs. 29a and 29b. An independent verification of these results was performed using the SAP program at BBN.

The maximum stress level of 645 N/mm^2 (93.5 ksi) is at the root near the leading edge on the suction surface. The high stress region of 90 ksi, however, extends only over a small portion of the suction surface (Fig. 29b) and so should not pose a problem for the planned test program. The permissible number of start/maximum speed/stop cycles is approximately 500, considering a notch condition (SCF = 3.5) at the juncture of the blade airfoil and the base shroud.

The tendency of the blade to untwist at the shroud location is small since there is only 1/2 degree difference in untwist between the shrouded and unshrouded NASTRAN results. significant load on the shroud, therefore, is the bending load due to the centilevered mass. The maximum shroud stress of 78.7 ksi is at the blade-shroud juncture, and is conservative in that the large fairing radius at the juncture was not included in the calculation. Because of the constraining effect of the mid-span shroud, the untwist of the blade at the shroud location is negligibly small. The untwist of the tip section calculated from the NASTRAN results is .36°, thus increasing the tip incidence from 2 to 2.4° at the design speed, a value well within the blade incidence design tolerance. However, radial growth of the shroud has not been accounted for and, if such growth occurs, undesirable increased tip incidence angle could result, due to the consequences of shroud sections "unlocking".





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FIG. 29a.

Pressure surface

Suction surface

FIG. 29b.

FIG. 29 NASTRAN STRESS ANALYSIS: FINAL ROTOR BLADE DESIGN

The magnitude of the stresses in the fan blade airfoil are acceptable for an experimental program. The computed stresses in this design exceed AVCO Lycoming practice for titanium blades for longtime service operation, but fall within acceptable limits for the planned experimental program.

5.7 Rotor Blade Vibration and Flutter

The avoidance of large amplitudes of resonant vibration of the rotor blades over the full operating range is necessary to ensure the structural integrity of the fan. The design procedure included an assessment of the natural frequencies of the rotor blade so that the forced vibration response is minimized, and the self-excited response is eliminated. The design goal for the minimization of forced vibration is ensuring that the rotor blade cannot resonate with the first three rotational orders of excitation due to possible inlet distortions. Although higher excitation orders will exist in the intake, it is considered that these levels will be minimal in the clean inflow expected in the acoustic test facilities and, thus, they will not generate significant resonant stress levels in the blade. The avoidance of self-excited blade vibration flutter is mandatory, since the associated stress levels usually lead to blade failure in a very short time. The two flutter phenomena that were considered in the design are subsonic positive stalled flutter at part-speed operation and supersonic unstalled flutter at design speed. criteria for avoidance of these flutter conditions are based on extensive experience by the engine manufacturers and are expressed in terms of a reduced velocity parameter: $u/b\omega$, where u =air velocity over the blade (m/sec), b = blade semichord (m), and ω = frequency of vibration in the flutter mode (rads/sec). The empirical design limit values for this parameter under positive stalled flow are 6.7 and 2.4 for the first bending and first torsion modes, respectively. The supersonic unstalled flutter design limit at first torsion frequency was:

$$\frac{u}{b\omega}$$
 $\left(\frac{M^2-1}{M}\right)$ < 1.05, where M = Mach number.

The coefficients are calculated at 3/4 span. (Since supersonic unstalled flutter usually occurs in vibration modes which are predominantly torsional, only this mode is considered.)

A free-standing blade, assumed fixed at the base, was used in the calculation of the resonant frequencies. The natural frequencies for the unshrouded blade are shown in the excitation diagram of Fig. 30. This design is clearly unsatisfactory since the natural frequency of the first bending mode has a second order resonance in the operating speed range. The stall flutter coefficients are 3.44 and 1.53 for bending and torsion, respectively. These values are within the safe limits which were established as design criteria. The supersonic unstalled flutter parameter is 1.16 and exceeds the safe upper limit.

A partspan shroud is required to raise both the first bending and torsion natural frequencies and avoid forced and self-excited vibrations (flutter). As a physical model, the shroud was assumed to restrict the blade motion to a uniform translation at three representative points.

The design analysis was checked by mounting two spare blades in a fixture which clamped at the root and partspan shroud locations. An acoustically coupled exciter was used to vibrate the blade so that the frequencies and mode shapes could be obtained. The comparison between the measured and theoretical static frequencies shown in Fig. 31, is considered good, especially in view of the unusual blade shape. The "measured" frequency line in Fig. 31 is actually the theoretical centrifugal stiffening line originating at the measured static frequencies of the first three modes.

Figure 31 shows the excitation diagram and calculated and measured frequencies for the final airfoil with the partspan shroud located at 64% of the span (201 mm radius). The first bending natural frequency has been raised so that it clears the first three excitation orders in the operating speed range. The fourth excitation order of the first mode, (e.g., four equally spaced front struts) however should be avoided. Based on the measured frequencies, the stalled flutter coefficients are 1.5 and 1.0 for bending and torsion, respectively. The supersonic unstalled flutter coefficient is .75. These values meet the design criteria. The excitation diagram shows that the torsion and bending modes are not coincident in the operating speed range. This ensures that the modes are decoupled.

Strain gauges will be used during the test program to ensure that safe steady and vibrating stress levels are not exceeded. In order to locate the strain gauges appropriately, a vibratory stress survey was conducted using strain gauges during the static vibration tests: Fig. 32 shows the results of this test, normalized for each mode. The vibratory stress distributions, shown as

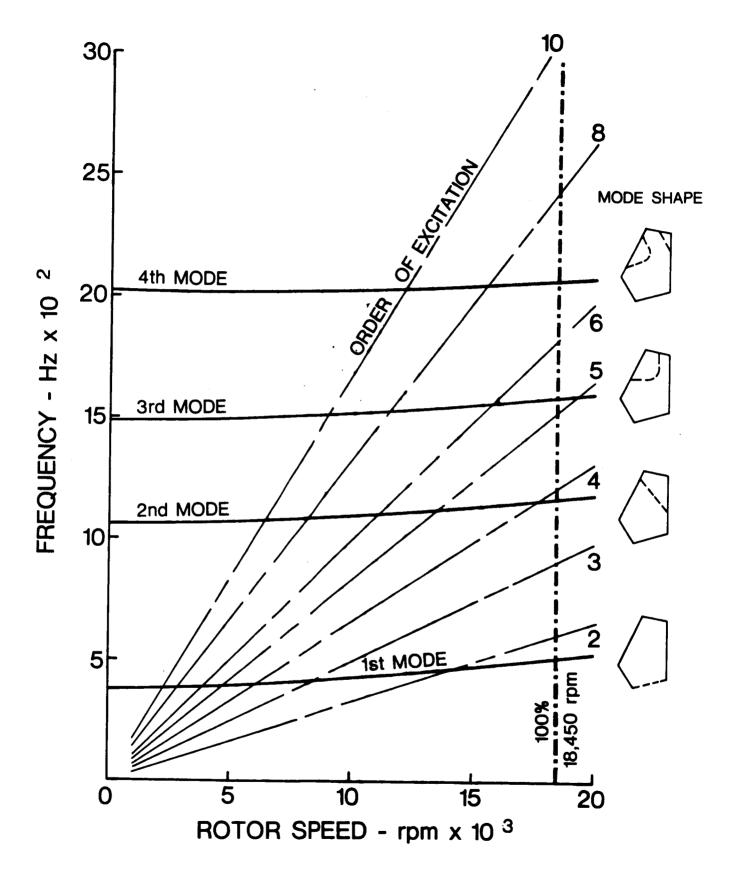
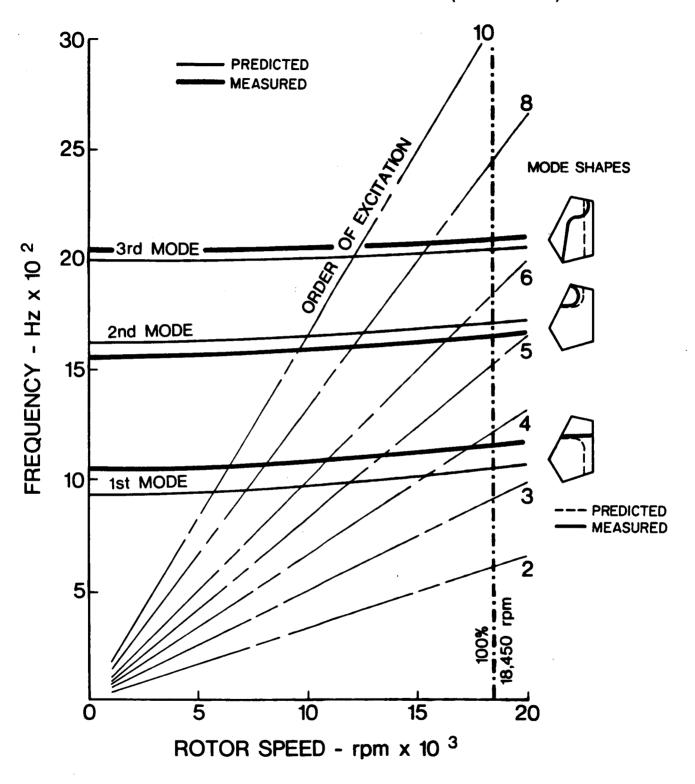


FIG. 30 RESONANCE DIAGRAM OF FINAL BLADE BEFORE SHROUD WAS ADDED.

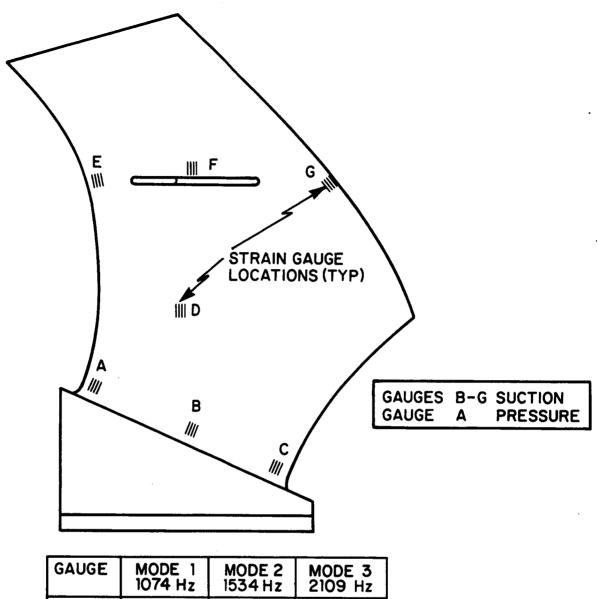
PARTSPAN SHROUD LOCATED AT 64% SPAN (r = 201 mm)



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FIG. 31 RESONANCE DIAGRAM OF FINAL (SHROUDED) BLADE.



GAUGE	MODE 1 1074 Hz	MODE 2 1534 Hz	MODE 3 2109 Hz	
Α	0.06	0.45	0.70)
B	0.06	0.38	0.22	
С	0.03	1.00	0.65	NORMALIZED
D	0.02	0	0.37	STRESS LEVELS
Ε	0.33	0.65	0.30	STRESS LEVELS
F	1.00	0.55	1.00	
G	0.13	0.18	0.43	

FIG. 32. MEASURED AND NORMALIZED STRESS DISTRIBUTIONS DURING STATIC VIBRATION TESTS ON BLADE S/N 17.

the combined steady and alternating stresses in the blade, are plotted for each mode in conjunction with calculated steady stresses at each gauge location used. Figure 33 shows the Goodman diagram for the blade material and the vibrating stresses measured in each mode proportioned for the most critical location. From this diagram it is seen that location 'F' is the most critical location in terms of combined stress in the first mode of vibration. Location 'C' is seen to be the most critical for the second and third modes of vibration. It is therefore recommended that strain gauges at positions 'C' and 'F' are used to monitor the steady and vibrating stresses during the rig running.

5.8 Attachment and Disk Analysis

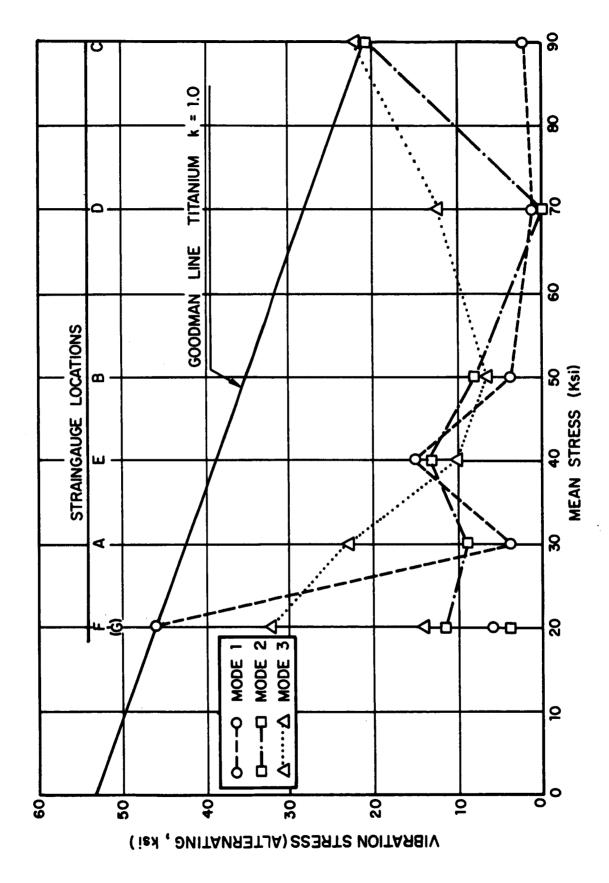
The fan disk stresses were computed by a Lycoming finite element program which evaluates the loading variation throughout the disk accounting for the effects of rotation, temperature gradients and elastic-plastic conditions.

Low cycle fatigue (LCF) life was evaluated for the significant regions, i.e., the disk serrations, the bolt holes and the disk bore, utilizing statistical minimum fatigue property data for Timkin 17-22AS material. The stress/strain ranges utilized in the life evaluation are the stabilized values corresponding to start/stop excursions to 18,450 rpm design speed.

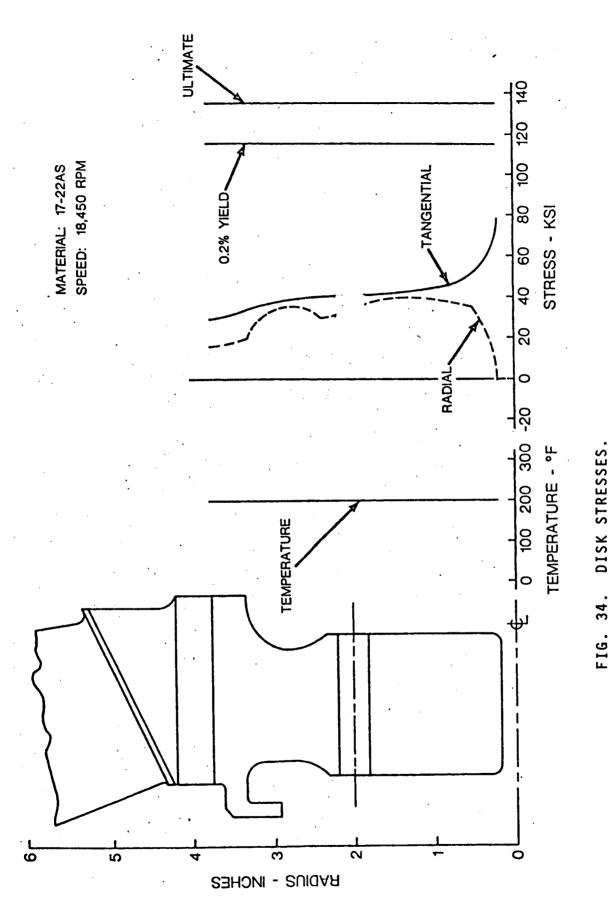
Stress concentration factors (SCF) were evaluated for those areas of the disk containing a high stress gradient, i.e., the serrations and bolt holes. This was accomplished by ratioing the peak stresses determined by finite element analyses with the nominal stresses in each of the two regions.

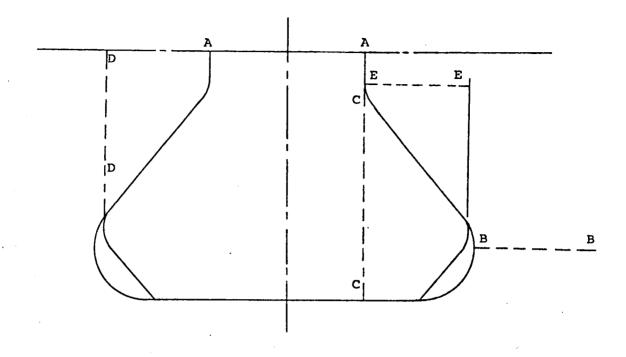
Nominal radial and tangential stress distributions for the fan disk are shown in Fig. 34, while the nominal stresses in the serration are shown in Fig. 35. The finite element models for the bolt holes and serrations are handled separately.

Stress distributions about the disk bolt-holes are given in Fig. 36 from which an SCF of 2.06 was calculated, so that the resulting LCF life is in excess of 100,000 "start/stop" cycles based on the material S-N data of Fig. 37. It has been concluded that the disk bore also has a calculated life of at least 100,000 cycles.



DETERMINATION OF CRITICAL VIBRATORY STRESS LOCATIONS. (Shrouded Blade). FIG. 33.





Part		Type of	Stress	Yield Strength	
	Location	Stress*	Level (ksi)	Disk	Blade
			(KSI)	(ksi)	(ksi)
Blade	A-A	Tensile	49.20	-	100,000
	C-C	Shear	21-00	-	50,000
Disk	В-В	Tensile	53.07	115,000	-
	D-D	Shear	17.36	57,500	_
	E-E	Bearing	61.03	_	_

*Includes C.F. and Bending Effect

Blade Material: Titanium 6AL-4V

Disk Material: 17-22AS

Temperature: 200°F Speed: 18,450 rpm

FIG. 35. DISK/BLADE ATTACHMENT STRESSES.

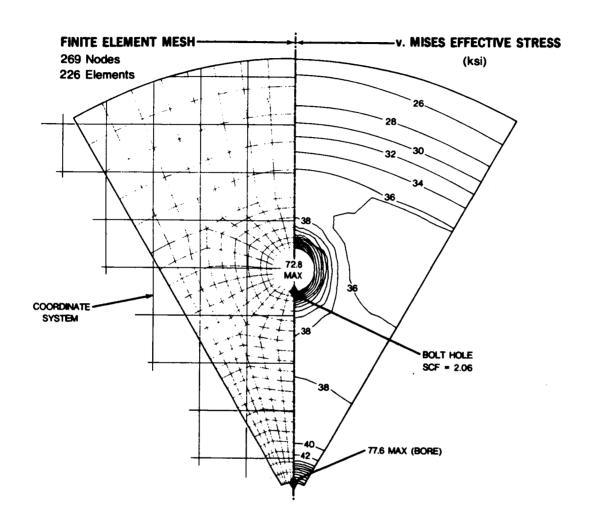
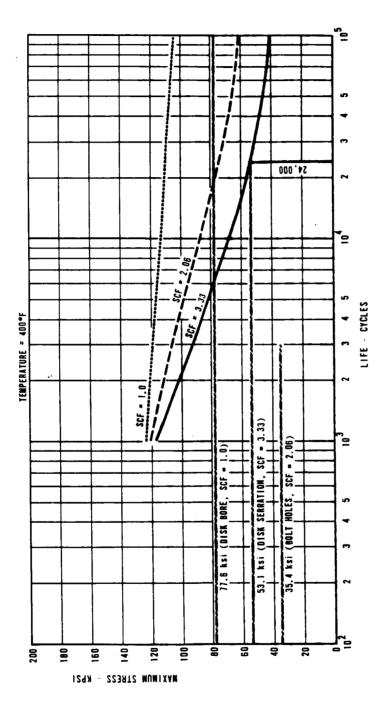


FIG. 36. DISK FINITE ELEMENT STRESS ANALYSIS.



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FIG. 37. MINIMUM DISK LOW CYCLE FATIGUE LIFE.

Blade root attachment stresses and corresponding material properties are also summarized in Fig. 35. The bending effects in both the root and tenon have been included. The axial width of the base-shrouded dovetail root was determined by a permissible bearing stress of 420 N/mm² (61.0 ksi). This is less than the compression yield limit, yet somewhat beyond the level at which fretting can occur under prolonged operation, but which should be satisfactory for a limited experimental program.

From a disk serration finite element analysis, the SCF was calculated to be 3.33 which is consistent with values measured from photo-elastic analyses of similar blade root configurations. The corresponding LCF life is 24,000 "start/stop" cycles based on the appropriate curve of Fig. 37. These fatigue lives are ample for the anticipated program of testing.

SECTION 6

DETAILED STATOR DESIGN

This section describes the detailed design of the fan stator which embodies the stator noise reduction concept described earlier. The stator uses vanes with varying sweepback angle to meet the criterion of a constant subsonic rotor wake trace speed along the stator vane span. The use of circumferential vane skew (lean) was avoided primarily to simplify the manufacturing problem. The stator vane number was chosen to cut off the radiation from residual sources due to end effects in the hub region of the vanes at blade passage frequency. The corresponding residual sources at the tip cannot be cut off because the spinning speed of wake disturbance pattern is supersonic at the tip.

To determine the proper vane sweep angle distribution, the rotor wakes were assumed to be convected with the mean flow. The spatial location of the wake centerline surfaces could then be computed from the mean flow properties by integration downstream from initial points on the rotor trailing edge. the rotor wake pattern spins fixed with respect to the rotor, it is possible to find leading edge lines whose shape is such that their point of intersection with the rotor wake centerlines travels at constant speed. Moving medium effects were taken into account in the actual calculation of a vane leading edge shape (see Appendix C for details). The trace speed was made constant and subsonic relative to the local flow velocity vector at all points on the vane span. The stator vane sweep distribution was designed to have an effective spanwise trace speed corresponding to a Mach number of 0.8 for the traveling load distribution.

The fundamental acoustical analysis which underlies the stator design concept is presented in Appendix C. In the remainder of this section, the methods for determination of the vane leading edge shape, and vane number are described, and the aerodynamic design considerations for the stator are reviewed.

6.1 Acoustic Aspects of Stator Design

The major noise producing mechanism of the stator is the interaction between the stator blades and the wakes shed by the rotor. This interaction causes fluctuating lift at the stator blades; the fluctuating lift in turn can be a potential source of noise. The fluctuating lift is restricted essentially near the stator blade leading edges (SBLE); this fact is made

abundantly clear from the analytical work of Filotas (Ref. 10). It is also well known (e.g., Lighthill, Ref. 15) that any fluctuating lift, whether at the leading or trailing edge, whether acoustically compact or not, whether in a stationary or moving acoustic medium, acts as a dipole source of sound.

However, irrespective of the nature of the sources (i.e., whether monopole, dipole or quadrupole, etc.), there are certain aspects of acoustics of stationary and uniformly moving media which need to be considered before approaching the specific task of stator design and related acoustic problems. Discussion of these fundamental aspects is provided in Appendices C.1 and C.2, and their application to the stator design of this fan is described below.

6.1.1 Criteria for non-radiation

Acoustic wavelengths at rotor blade passage frequency are small compared to the stator blade span. In this case, the criterion for non-radiation due to unsteady forces is that the trace phase velocity of the force disturbance be subsonic relative to the local gas flow. Skewing, or sweeping of the stator blade, increasing the separation between rotor and stator, and shaping the rotor blade are techniques which can be used to reduce the phase trace speed.

Proper modification of leading edge profiles can reduce the phase trace speed along the leading edge and also the relative angle between that velocity and the local flow. Both effects are important as it is the trace velocity relative to the local gas-properties flow which must be kept subsonic.

Each individual wake shed by a rotor blade suffers a lag in the circumferential direction. The net effect of this lag on the nature of impingement of the wake on an unswept SBLE is that the wake hits the SBLE at the hub first and the impingement process propagates radially outwards towards the SBLE tip with a spanwise varying phase or trace velocity $c_0(r)$. Sweeping back the SBLE enhances this phase lag effect, in the sense that the spanwise trace velocity of wake impingement is reduced. A criterion along the lines of Eqs. (C.62) and (C.63) is used to

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guarantee that the wake trace Mach number $m_{\rm O}$ is less than $m_{\rm U}$ everywhere along the swept back SBLE. The nature of the wake phase lag and the calculation of the SBLE sweep angle is described more fully below. The successful analysis of a rotor wake tracing along the leading edge of a stator requires understanding of a set of transformations between stator-fixed coordinates and moving medium coordinates. The derivation of the trace velocity in stator-fixed coordinates, and subsequent Gallilean transformations to gas-fixed coordinates is given in Appendix E.

6.1.2 Estimate of rotor viscous wake

Estimates of the magnitude of the rotor viscous wake at the leading edge of the stator have been made. The method of estimation involved modeling the rotor blade wakes as the wakes behind isolated airfoils. The method is somewhat crude, as it ignores the interference between wakes and the axial pressure gradient. The variation of angle of attack at the stator which results from the estimated velocity fluctuations is as much as 10 degrees from the mean. Experience with axial flow turbomachinery wakes indicates that the estimated rotor wake amplitudes at the stator leading edge are likely to predict higher resultant angle of attack fluctuations than will exist in the actual rotor wake. This is due to the higher rate of decay of rotor blade viscous wakes in turbomachines when compared to isolated viscous wakes in free flow (see, for example, Lakshminarayana and Raj; Ref. 16).

6.1.3 Computation of rotor wake distortion

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Contours of constant phase for rotor wakes at different axial locations were computed by use of a stepwise integration of the phase lag of the wake relative to a point in the rotor, as a function of radius. Cylindrical helical flow was assumed (radial flow velocity was assumed to be zero). Axial and tangential velocities used in this calculation were provided by AVCO's aerodynamic design program (Appendix A). Contours of constant phase calculated at several stations downstream of the rotor are shown in Fig. 38 (see Fig. 14 and Appendix A for Station Locations). Contours of constant phase versus axial location on cylindrical surfaces, shown in Fig. 39 and contours of constant phase in the axial/radial plane, shown in Fig. 40, were derived by cross-plotting from Fig. 38.

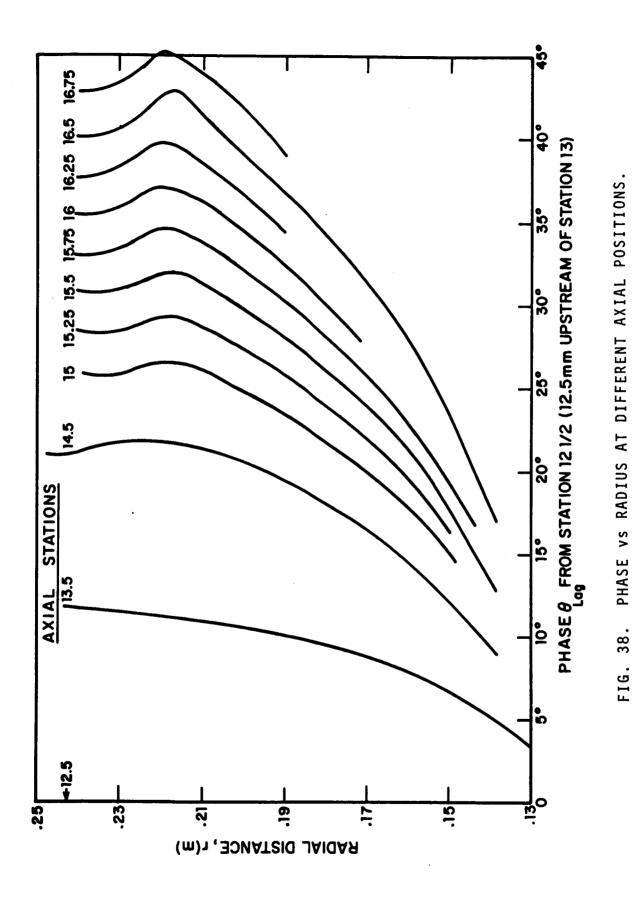
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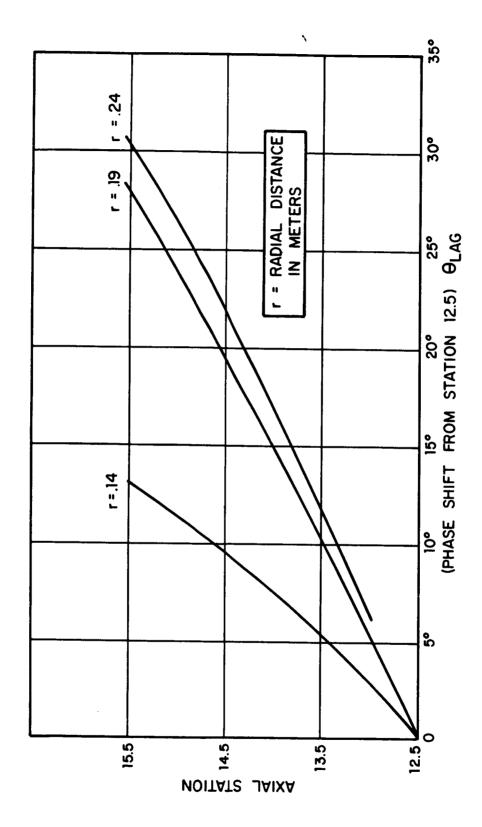
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PHASE vs AXIAL LOCATION ON CYLINDRICAL SURFACES. FIG. 39.

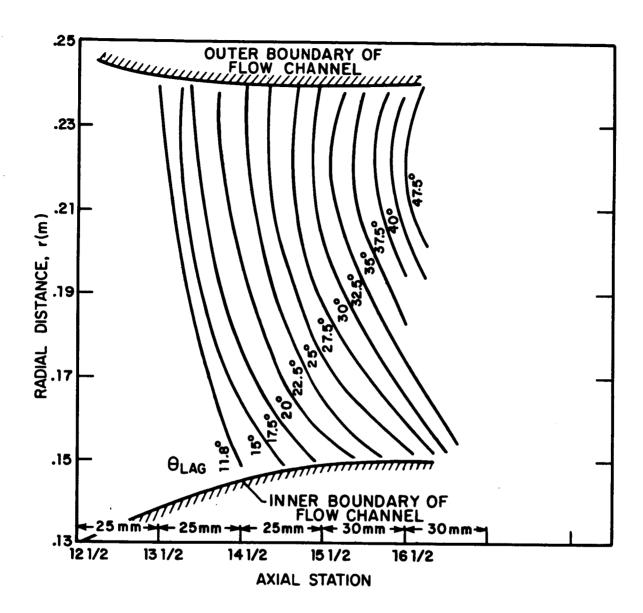


FIG 40. CONTOURS OF CONSTANT PHASE IN Y-Z PLANE.

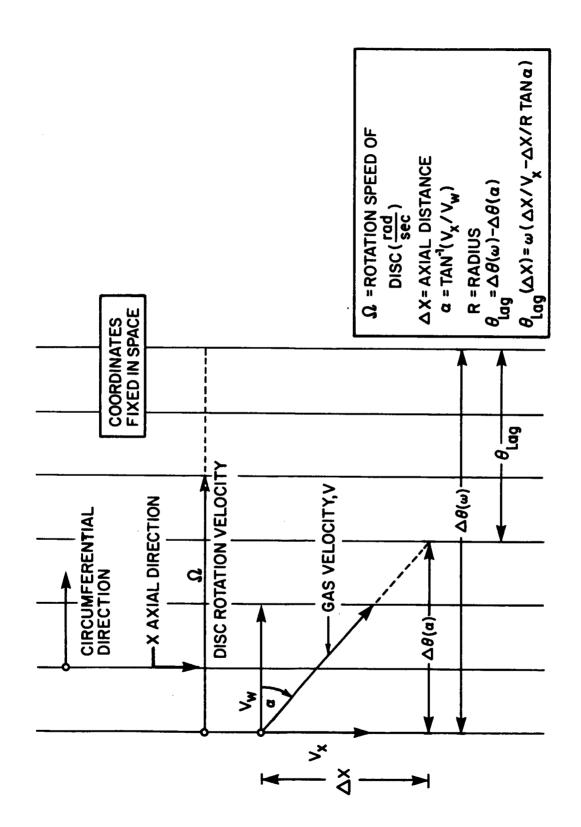
Figure 41 illustrates, in stationary coordinates, the flow and blade motion geometry and the equations used in the calculation of the constant phase contours. The two terms $\Theta(\alpha)$ and $\Theta(\omega)$ in the calculations represent the angular translation of a fluid particle and the angular rotation of the rotor, respectively, in the time required for the fluid particle to flow from axial Station I to Station 2. The trace velocity, relative to local flow, for a number of constant-sweep-angle stators is shown in Fig. 42 The very high trace velocities near the tips result from reduced wake "windup" in that region, thus illustrating the limited effectiveness of constant angle swept stators.

6.1.4 Mach .78 leading edge stator

A blade leading edge sweep profile for trace speeds less than Mach 0.8 was developed for the final rotor and flow path design using an iterative method to achieve a nearly uniform trace velocity. The blade has a minimum sweep angle of 25 degrees at approximately 1/3 of the span from the root. Sweep at the root and tip are 30 and 40 degrees, respectively. Figure 43 shows the sweep profile as well as the trace and acoustic speeds as a function of radius.

The calculations assume a rotor blade reference axis at the axial location 12-1/2 mm. forward of the root at Station 13. It also assumes a stator leading edge which is radial, when projected in the r, θ plane, and has its root 12.5 mm. downstream of Station 15.

The inflow-induced radiation from an array of such variably-swept stator vanes is now restricted to the tip regions when discontinuities occur. The limitation of such radiation depends upon proper choice of the number of swept leading edge stators, which in turn depends upon the rotor blade number, rotation speed, and moving medium acoustical considerations. The computation of vane numbers is discussed below.



PHASE LAG, θ_{1ag} , RELATIVE TO POINT OF ORIGIN FIXED ON ROTATING ACTUATOR. FIG. 41.

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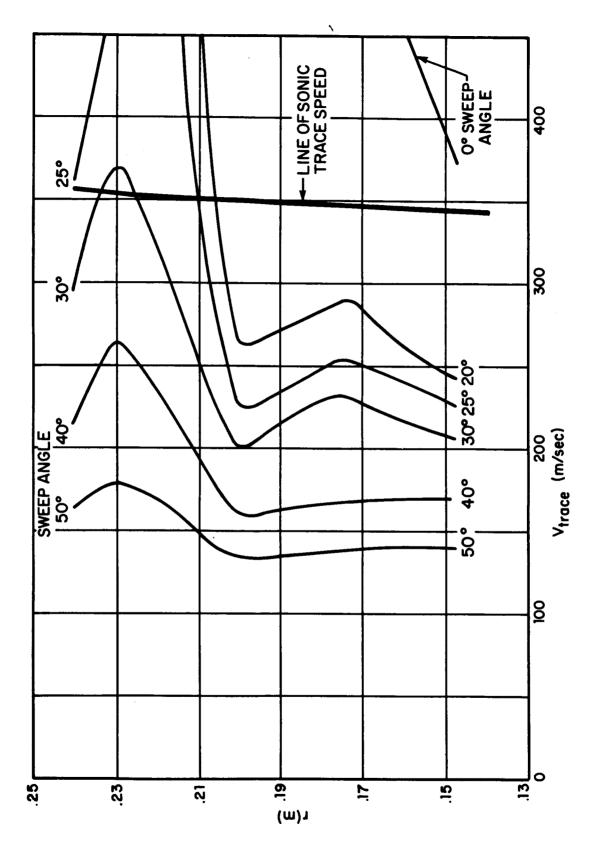
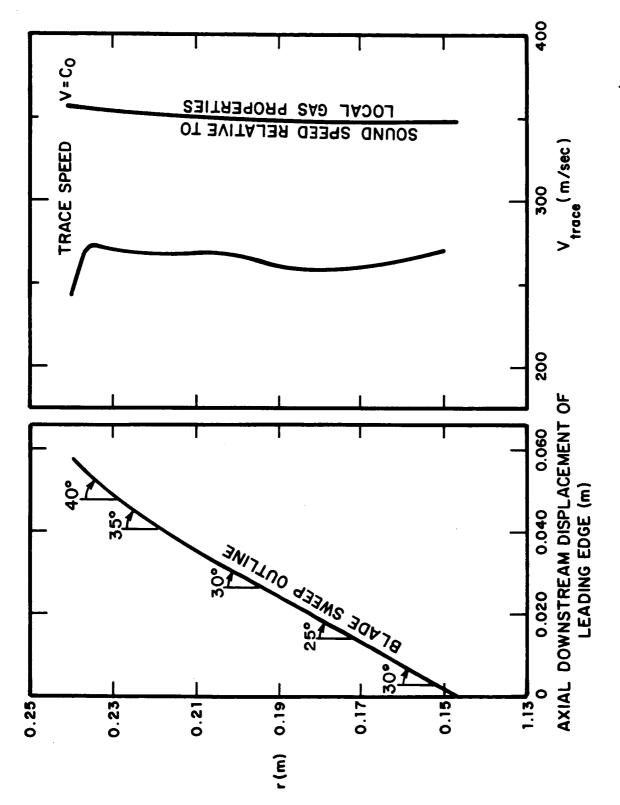


FIG. 42. TRACE VELOCITY FOR DIFFERENT SWEEP ANGLES.



MACH .78 LEADING EDGE PROFILE (FINAL VANE DESIGN). FIG. 43.

6.2 Analysis For Determination Of Number Of Stator Blades

This section determines the appropriate minimum number V of stator blades so that the acoustic noise radiated from the stator at the ${\bf rotor}$ blade passage frequency ${\bf f}_{\bf r}$ is minimized.

Since the swept back SBLE derived in the above section is of finite extent, the end effects at the SBLE hub and tip from the wake/SBLE interactions remain as potential sources of noise. The aim here is to seek a partial circumferential cancellation of these end sources. Thus, two discrete circumferential arrays exist, one at the SBLE hub and the other at the SBLE tips. Since the circumferential phase velocity c is higher at the tip than at the hub, one concentrates on the discrete circumferential array made up of uncancelled sources at the SBLE tips. Also, as discussed in Appendix C, the discussion of a discrete array must be limited to only one frequency $\omega_{\rm O}$. Choosing ω to correspond to the fundamental rotor harmonic, (i.e., to the rotor blade passage frequency $f_{\rm P}$), one obtains

$$\omega_{\Omega} = \Omega B \quad , \tag{17}$$

where Ω is the shaft rotation in radian/sec (Ω ~ 1940 rad/sec) and B is the number of rotor blades (B = 28). The blade passage frequency f_r is given by

$$f_r = \frac{\omega_o}{2\pi} \approx 8600 \text{ Hz} \tag{18}$$

For the circumferential phase velocity c

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$$c_0 = \Omega r_t$$
 , (19)

where r_t is the radius at the stator tip (r_t $\stackrel{>}{_{\sim}}$ 0.24m $\stackrel{>}{_{\sim}}$ 0.79 ft). The corresponding Mach number m_o is therefore given by

$$m_o = \frac{c_o}{c} \approx 1.27 \tag{20}$$

With reference to results of Appendix C.2, (Eq. C.60), m_1 is the Mach number of the gas flow parallel to the array, and m_1 is the gas Mach number normal to the array. Since the array under consideration is oriented circumferentially, the gas Mach number m_1 in the circumferential direction plays the role of m_1 and the gas Mach number m_2 in the axial direction plays the role of m_2 ; the radial gas Mach number, normal to the duct walls, is to a good approximation zero. Thus, we have

$$m_1 \equiv m_c \approx 0.353 \tag{21}$$

$$m_{r} \equiv m_{a} \approx 0.582 \tag{22}$$

Note that $\rm m_c$ is directed the same way as the shaft rotation Ω or the phase Mach number m . Hence, first one would like to find from Eqs. C.62 and C.63 whether $\rm m_0 < \rm m_u$ - one of the two necessary conditions for no radiation to occur. Substituting the quoted values in Eq. C.62, one finds that

$$m_0 < m_u$$
, for $\frac{\pi}{2} < \alpha \leq \frac{\pi}{4}$, (23)

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in other words, the condition for no radiation is satisfied for angles α that are sufficiently remote from the axial direction.

In order to satisfy the second condition for (partial) cancellation of radiation from the fundamental rotor harmonic, it is required that

$$\frac{2\pi}{d_{t}} > 2 \frac{2\pi}{\lambda_{r}} \frac{(1-m_{a}^{2}\cos^{2}\alpha)^{1/2}}{(1-m_{c}^{2}-m_{a}^{2}\cos^{2}\alpha)}, \qquad (24)$$

where d_t is the circumferential spacing between two adjacent SBLE tips. The spacing, d_t , is related to the number of stator blades by the relation

$$d_{t} = \frac{2\pi r_{t}}{V} \tag{25}$$

The right hand side of Eq.(24) is the radiation span $(k_a - k_a)$ obtained from Eq. 0.60, where $2\pi/\lambda_p$ has been of one substituted for k_a , λ_p being the acoustic wavelength at frequency fr. o Taking sound speed c in the gas to be about 365 m/s (1200 ft/sec), the wavelength at blade passage frequency is

$$\lambda_{\mathbf{r}} = \frac{\mathbf{c}}{\mathbf{f}_{\mathbf{r}}} \approx 0.14 \text{ ft} \approx 0.043 \text{ m}$$
 (26)

Thus, substituting Eq. 25 in Eq. 24, the velocity is

$$V > \frac{4\pi r_{t}}{^{\lambda}r} \frac{(1-m_{a}^{2}\cos^{2}\alpha)^{\frac{1}{2}}}{(1-m_{c}^{2}-m_{a}^{2}\cos^{2}\alpha)}$$
 (27)

Since the first necessary condition (Eq. 23) is satisfied only for a restricted range of angles α , it would not pay to find the maximum possible value of V for arbitrary α . Instead, Eq. 27 is evaluated for α = $\pi/4$ (as α goes from $\pi/2$ to $\pi/4$ to 0, V evaluated from Eq. 27 increases), and the result is

$$V \ge 92 \tag{28}$$

One can now examine the application of traditional analyses (e.g., Tyler and Sofrin (Ref. 11)) of noise generated by rotorstator interaction, the anlysis that is used primarily for low-speed compressors (i.e., analysis is based on stationary medium acoustics) that involve subsonic circumferential phase speeds (i.e., $\Omega r_t < c$) and are acoustically compact (i.e., $d_t < \lambda_r/2$).

An arbitrary component (say, the predominant component of wake velocity deficit pattern that generates fluctuating lift at SBLE) $a(x,r,\theta,t)$ of rotor-generated flow field near the stator may be decomposed into circumferential harmonics as follows

$$a(x,r,\theta,t) = \sum_{n=-\infty}^{+\infty} A_n(x,r) e^{i[nB(\theta-\Omega t)]}, \quad (29)$$

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where x and r are the axial and radial locations (and for our case of interest denote the locations of SBLE tips) and θ is the circumferential angle.

The noise sources (in particular, the fluctuating lift ℓ generated at SBLE tips) at the stator due to the nth rotor harmonic may then be viewed as composed of a sum of stator/rotor harmonics mn. A typical interaction harmonic $L_{mn}(s,r,\theta,t)$ may be written as

$$L_{mn}(x,r,\theta,t) = L_{mn}(x,r) e \qquad (30)$$

where

$$m = nB + kV , \qquad (31)$$

and where k can assume arbitrary integral values (positive, negative or zero).

The circumferential phase velocity $c_{\rm o}({\bf r})_{\rm mn}$ associated with Eqs. 30 and 31 can be written as

$$c_{o}(r)_{mn} = \frac{nB\Omega r}{B+kV}, \qquad (32)$$

Similarly, from Eq. 19, the circumferential phase velocity for all the rotor harmonics n is Ωr and assumes the value Ωr_t at the SBLE tips. This same value is recovered for the interaction modes from Eq. 32, for the stator fundamental mode, i.e., for the case k=0.

Figure 44 depicts the situation in terms of these rotorstator interaction harmonics. The harmonics lie at the intersections of vertical lines passing through the nB axis for n = 0, ± 1 , ± 2 ... and horizontal lines passing through the m axis for k = 0, ± 1 , ± 2 ... The rotor fundamental tone occurring at the blade passage frequency fr (see Eqs. 17 and 18) corresponds to vertical straight lines passing through $n = \pm 1$ (i.e., $nB = \pm 28$). Similarly, nth rotor harmonic corresponds to frequency nfr. The fact that attention was turned to the stator fundamental harmonic at frequency fr (see Eqs. 19 and 20) means that the k = 0 stator mode was examined at f_m . From Eq. 23 , one finds that this stator fundamental harmonic barely escapes radiation. From Eq. 32, it can be seen that the same situation applies to stator fundamental harmonics (i.e., k = 0modes) for all rotor harmonics (i.e., arbitrary n). Thus, the straight line in Fig. 44 passing through these k = 0 modes separates the radiating and non-radiating harmonics.

The criterion of Eq. (24) was applied to prevent the next candidate stator harmonics ($k = \pm 1$ modes for $n = \pm 1$) from radiating at the blade passage frequency f_r (only). The straight line joining these k = -1, +1 modes thus also separates the radiating and non-radiating harmonics. The flow-induced assymmetry in radiation span along wavenumber is reflected in Fig. 44 by assymmetry of radiating and non-radiating harmonics around m and nB axes. Incidentally, stator harmonics lying in the upper right and lower left quadrants of the m, nB plane possess circumferential velocities that are oriented in the same direction as shaft rotation Ω , and the harmonics lying in the upper left and lower right quadrants possess velocities that are oriented in the direction opposed to Ω .

Finally, note that the relatively high number V of stator blades indicated by Eq. 28 may cause design problems of aerodynamic nature. For example, relatively high solidity at the hub, particularly for the scale model fan, is unacepceptable. Therefore, a compromise number of 59 was selected for V. Such a choice ensures circumferential cancellation at the SBLE hub, but not at the tip. In other words, with reference to Fig. 44, k = ±1 modes would radiate from the SBLE tips.

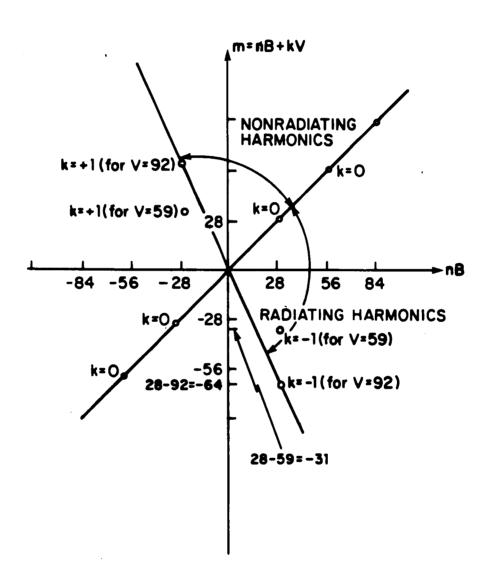


FIG. 44. SKETCH OF RADIATING AND NON-RADIATING ROTOR/STATOR INTERACTION OF HARMONICS (nB,m).

6.3 Stator Aerodynamic Design Considerations

The stator is characterized by a backward leading edge sweep varying from 25 to 40° in the meridional plane. The axial spacing between rotor trailing edge (TE) and stator leading edge (LE) varies approximately from two to three rotor hub chords along the span.

A minimum number of 59 blades was specified by acoustic considerations. This, in conjunction with a tentatively selected hub cascade solidity $\sigma_{\rm hub}$ = 2, resulted in a chord length of approximately 30mm.

The meridional contour of the stator was shown in Fig. 14. Radial station 17 crosses the leading edge, 19 the trailing edge, and 18 crosses both the leading and trailing edges. Radial equilibrium along those stations is markedly influenced by the varying degree of stator turning, resulting in peculiar tangential velocity distributions that have been input in R-121, together with the corresponding total pressure loss distributions. The flow conditions from rotor exit station 13 to stator inlet stations 16, 17, 18, and 19, have been calculated according to constant rotor exit momentum V_u·r specified along the streamlines.

The meridional flow pattern (Fig. 14) shows the radial streamline shifts induced by the swept back stator configuration, especially in the hub region, where $V_{\rm u} \cdot {\rm r}$ is large and has a strong effect on radial equilibrium. In the axisymmetric flow case, the streamlines approaching the leading edge are deflected inboard. Looking at the lower portion of station 17, the flow at the hub section has already undergone the major part of its turning, and $V_{\rm u} \cdot {\rm r}$ thus increases markedly from the hub to the 30% streamline on that station. This substantial departure from free-vortex flows generates an increase of the axial velocity component toward the hub and a corresponding increase of the mass flow density $\rho V_{\rm x}$, in turn resulting in inboard streamline shifts between leading edge and station 17. This characteristic pattern is found along the entire span, but disappears gradually toward the tip section because of the decreasing value of the $V_{\rm u}^2/r$ -term.

The meridional streamline curvature term V_m^2/R_c has a strong effect in this flow field region. The determination of R_c however is very approximative even with the spline-on spline procedure

used in R-121, so that the interpolated values of the flow conditions at the stator leading edge cannot be expected to be smooth. Fig. 45 shows the radial distribution of the inlet angles α_3 determined by $V_{\rm x}$ -interpolation and constant $V_{\rm u} \cdot {\rm r}$ along the streamlines, and the smooth distribution assumed for blading design. The maximum smoothing error does not exceed 1-1.5°, which is well within the accuracy that can be expected from the axisymmetric analysis.

Figure 45 also shows the stator exit flow deviation angles δ calculated with Carter's formula [Equation (8), circular mean camber lines]. The blade sections are stacked with the leading edge in a meridional plane. All profiles were set at 0° nominal incidence. Table 6 lists the profile data defining plane sections perpendicular to the radius in the leading edge plane.

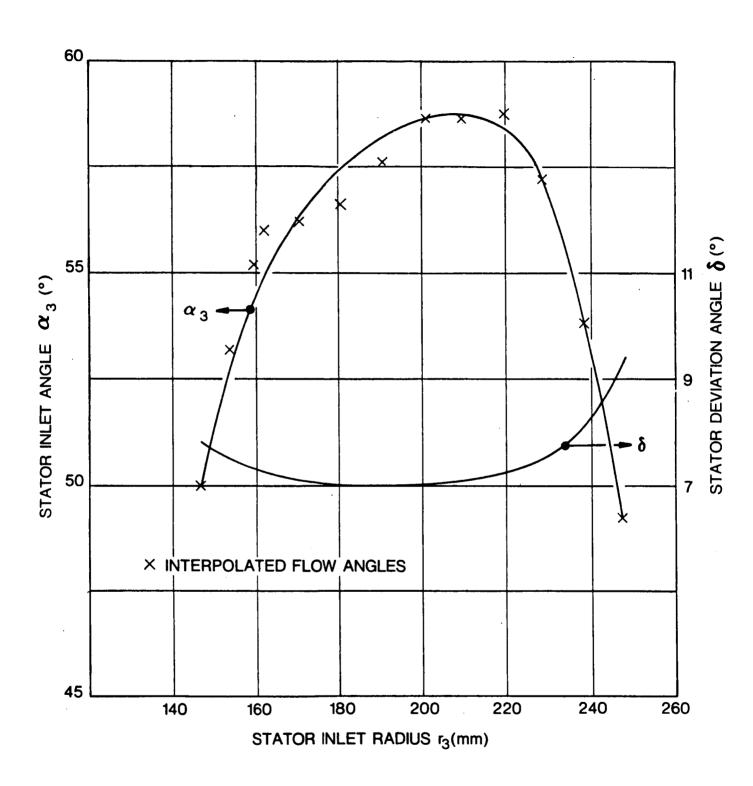


FIG. 45. STATOR INLET FLOW AND DEVIATION ANGLES.

TABLE 6. STATOR BLADE DATA

	59 Blades	les			:			(P1	(Plane Sections)	tions)
Section Radius (mm)	147	154	160	171	191	210	228	236	242	248
Mean Camber Angle • (°)	08.74	06.44	43.00	1.90 43.00 40.80 38.90 38.40 40.00 43.1	38.90	38.40	40.00	43.1	46.50	50.20
Setting Angle γ (°)	73.90	75.05	75.80	75.05 75.80 76.70 77.55 77.90 77.50 76.55	77.55	77.90	77.50	76.55	75.45 74.30	74.30
Chord Length c (mm)	31.3	31.3	31.5	31.85	33.05	34.80	36.90	37.9	38.75	39.60
Diffusion Factor D	396:								-	.288
Rel. Thickness v (%)	0.9	ı	ı	ı	1	1	ı	ı	ı	0.9
Axial Coordinate of L.E. $z_{\rm L}$ (mm)*	0	4.1	7.8	14.0 25.1	25.1	35.0	47.4	53.7	58.7	64.0

*Leading Edge swept back in a meridional plane.

SECTION 7

COMMENTS ON RESIDUAL NOISE SOURCES AND NOISE LEVELS OF THE SWEPT ROTOR AND STATOR FAN STAGE

The object of the Low Source Noise Fan Program is to design fan components for minimal noise generation. This has been done by first using physical models for each of the component noise mechanisms, and calculating the appropriate parameters from the particular baseline fan design, then modifying the component geometry to minimize noise generation. All sources of noise cannot be eliminated, and indeed all sources have not been attacked in this study.

7.1 Residual Sources for a Fan outside the Laboratory Environment.

As has been previously discussed in detail, the compound sweep required on the rotor blades for structural reasons will lead to a conical shock at the sweep reversal point. However, in some future fan designs, the location of the sweep reversal point at a radius less than that at which the critical relative Mach number occurs will eliminate the source of noise. Rotor discrete frequency mechanisms which cannot be eliminated include the so-called Gutin noise sources associated with steady loads and thickness noise. However, the non-radial blading may cause these mechanisms to excite high order duct modes and thus reduce the radiation to the far field. Rotor broadband mechanisms are relatively poorly understood quantitatively (in the absence of inflow turbulence), and thus are difficult to attack at Shock/turbulence interaction in the channel may cause the source. some forward radiated noise, and quite likely causes aft-radiated broadband noise.

Stator noise mechanisms are much better understood and can be attacked with much more confidence than some rotor mechanisms. The uncancelled tip radiation (calculated in Appendix C) is the only discrete-frequency mechanism inherently associated with the subsonic trace speed swept stator concept, assuming that the rotor wake field can be accurately specified. Stator broadband mechanisms not attacked by the swept leading edge include vortex shedding and flow separation at the trailing edge.

Other broadband noise from an installed fan comes from the exhaust jet and duct boundary layer turbulence interaction with the lip of the fan duct.

7.2 Prediction of Noise Levels and Noise Reduction of the Swept Rotor and Stator Fan.

Despite intensive research efforts in the past twenty years which have led to a good understanding of noise mechanisms and scaling laws, the ability to predict fan noise for an arbitrary design on a component-by-component basis is quite limited. For conventional fans, useful semi-empirical correlations of data have been made using scaling laws which are based on assumed mechanisms. Thus, for conventional fans, one can predict within a few dB the expected sound power and directivity. However, the applicability of those correlations to a fan of unconventional component design is doubtful.

For the subject fan design, the prediction of residual noise from the rotor requires the knowledge of the strength of the conical shock upstream of the rotor, which is not presently known due to the cessation of activity on the 3-D compressible flow program. The stator discrete noise has been calculated directly for basic principles and is presented in Appendix C.

However, the main noise source of interest, rotor multiple pure tones cannot be reliably estimated without detailed information on shock structure and duct propagation characteristics. In the interest of providing an estimate of the benefits of eliminating MPT noise, a computer program published by Burdsall $et\ al.$, (Ref. 20) was exercised (see Appendix D for details). The results summarized below for a full scale (a 40,000 lb thrust) counterpart of the 20 inch fan built in this program, show that elimination of the shock-generated MPT's reduces the overall and perceived noise levels by 4-6 dB, and reduces the tonal content in the 1/3 octave band containing the blade passage frequency by about 10 dB.

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TABLE 7. ORDER-OF-MAGNITUDE EMPIRICAL ESTIMATE OF NOISE LEVELS FROM FULL SCALE SINGLE STAGE FAN.

Spectrum Component	Overall PWL (dB re 10 ⁻¹² w)	0ASPL*(@150') dB(re 2x10 ⁻⁵ N/m²)	PNL*(@150!)
M.P.T.(conven- tional blades)	152-154	104-106	117-119
B.P. Tone	143	95	108
Broadband Mech- anisms	150	103	114
TOTAL with MPT's without MPT's	153.5-155.2 150	105.3 - 107.2 103	118-120 114

^{*}Valid in the forward-radiated direction at azimuths from 40-80° from fan axis (± 3 dB); to scale to greater distances, subtract 20 log r/150, where r is distance in feet.

SECTION 8

MECHANICAL DESIGN ASPECTS AND FACILITY INTEGRATION

The fan rig is built to conventional standards and is designed to interface with the NASA W-2 and W-8 test facilities. The W-2, acoustic facility is arranged for the measurement of forward and rearward radiated noise; thus, the rig casings have flanges at both ends which mate with the facility mounting flanges. The manner in which this is accomplished is shown schematically in Fig. 12. In the reverse flow mode, for backward radiated noise, an additional flow path adaptor supplied by NASA and not shown in Fig. 12 is fitted to the fan outlet flanges. In the W-8 facility the fan is mounted on its rear flange with the flow entering from the bellmouth. All detailed performance measurements of the fan will be made in the W-8 facility.

A flow path adaptor fits over the facility bearing housing to control the fan outlet flow and into this adaptor is fastened the inner shroud of the stator vane assembly. The outer shroud of the stator vane assembly is located in the fan casing and provision is made for axial adjustment of the stator by relocating the spacers at the inner and outer shrouds. The fan outer casing is split in the vertical plane for assembly purposes. The section of the outer casing in the area of the blade tips is relieved and an abradable shroud lining is installed to prevent blade tip damage in case of tip rubs. Figure 46 shows the engineering cross-section of the fan which details all the major components. Strain gauges will be applied to the rotor blades, the wires being led down the front and rear faces of the disc. W-2 facility the slipring is installed at the driven end of the rig shaft and, thus, the strain gauge wiring will pass down the length of the hollow shaft. When the fan is running in the W-8 aerodynamic facility, the strain gauge wiring will be led forward through the driveshaft adaptor which is fitted in place of the spinner support cone. A static fairing is installed over the slipring to provide a smooth flow profile into the fan, in place of the spinner.

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FIG. 46. DETAILED CROSS-SECTION OF FAN RIG.

SECTION 9

CONCLUDING REMARKS

A research program was undertaken to try to demonstrate that source noise reduction concepts which are based upon full and rigorous application of fundamental aeroacoustic principles can be implemented on turbofans in the currently-operating range of tip speeds and pressure ratios, utilizing the existing design and manufacturing capabilities of the aircraft engine industry, without serious compromise of the noise reduction concept.

The subsonic leading edge rotor blade concept has significant potential as a practical solution to rotor-generated noise due to its inherent lack of sensitivity to off-design-point operating conditions, and the large family of detailed edge and generating surface contours available for fans in various speed ranges. The aerodynamic behavior of subsonic leading edge rotors in supersonic absolute inflow velocities is largely unknown at this point in time. However, it is believed that the characterizations of such flow fields, to the extent necessary in developing actual engines using the subsonic leading edge rotor principle, would at this time require considerably less effort than has been expended historically in understanding aerodynamic behavior of conventional rotors.

The subsonic trace speed stator vane concept can be implemented through application of moving medium acoustic principles and a knowledge of the details of the rotor wake field, the lack of the latter being a current limitation. However, the subsonic trace speed concept can, in principle, be successfully implemented by use of conservative assumptions about the rotor wake field.

APPENDIX A

COMPUTER LISTING OF AEROTHERMODYNAMIC PARAMETERS FOR FINAL ROTOR, STATOR & FLOW PATH DESIGN

APPENDIX A: DETAILED AEROTHERMODYNAMIC DATA

This appendix contains a computer listing of the aerothermo-dynamic data for the final design of the fan.

The first four pages, A-6 to A-9 are input data to AVCO Lycoming Program R121 at various axial stations. All units in the SI system and headings on the columns are self-explanatory. The three parameters in the left hand column are:

TOT PRESS = Total Pressure Ratio

TOT TEMP = Total Temperature Ratio

The remaining pages are detailed output at the various axial stations, the non-obvious terms of which are defined below.

Coded Term	Meaning	Units
A STATIC	ambient sound speed	m/sec
A TOTAL	sound speed based on total temperature	
ALPHA BAR	\sin^{-1} (Vm/V)= angle of flow made by V in tangential direction measured on a cone	degrees
ALPHA	\sin^{-1} (Vx/V) = angle made by projection of absolute velocity vector (on a cylinder)	degrees
BETA	angle that the relative velocity vector makes with a cylinder	degrees
V	absolute velocity	m/s
VM	meridional component of V	m/s
VR	radial component of V	m/s
S-VALUE	radial length measured along a station cut (origin at hub)	m
% SPAN	percent radial distance compared to full span measured from hub	
VX	axial component of absolute velocity	m/s
VU	tangential component of absolute velocity	m/s
W	relative velocity	m/s
WU	tangential component of relative velocity	m/s
MV	Mach number of absolute velocity V	
MVX	Mach number of VX	
MVM	Mach number of VM	

Coded Term	Meaning	Units
R-ADC	streamline radius of curvature in meridional plane	m
RHO	fluid density	Kg/m³
ROTOR EFF	polytropic rotor efficiency	
S-VALUE	radial length measured along a station cut (origin at hub)	m
STAT PRESS	static pressure	bars
STAT TEMP	static temperature	° Kelvin
TO/TO (TO/TO)T }	Similar definitions for the stagnation temperatures.	
TOT PRESS	total pressure ratio	
TOT TEMP	total temperature	° Kelvin
U	rotational speed	m/s
V	absolute velocity	m/s
VM	meridional component of V	m/s
VR	radial component of V	m/s
VU	tangential component of absolute velocity	m/s
VX	axial component of absolute velocity	m/s
W	relative velocity	m/s
WU	tangential component of relative velocity	m/s
X-VALUE	axial location of station re:origin (station 4)	m

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Coded Term

Meaning

Units

% AREA

% of annulus area taken up by a stream % tube from the preceding area to that where % AREA is indicated

FLOW CONDITIONS IN MULTI-STAGE AXIAL COMPRESSORS AND TURRINES

AVCC-LYCOMING PROGRAM RIZI N. REPNSTEIN

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0.0	3 AD TUS	0.0610	1.013 288.00 0.0	PADIUS	.0707	1.013 288.00 0.0	2 AD TUS	0.0780	1.013 288.00 0.0	RADIUS	0.0860	1.013 288.00 0.0	340108	0.0940	1.013 288.00 0.0	RADIUS	0.1020	1.013 288.00 0.0
ΛΩ	HUB 3	0	252	HUB. P.	ŏ	TOT PRESS TOT TEMP	4.1JR 2	O	TOT PRESS	A RUH	0	TOT PRESS TOT TEMP	₩ ₩	0	TOT PRESS TOT TEMP	F BUH	0	TOT PRESS TOT TEMP

		1.013 288.00 0.0			1.661			1.661 347.60 0.0		!	1.661 347.60 0.0			1.661	i		1.661 342.40 127.50
!		1.013 288.00 0.0			1.661	٠		1.661 345.10 0.0	:		1.661 345.10 0.0			1.661 341.40 0.0			1.561
		1.017 288.00 0.0			1.661 342.80			1.461 342.80 0.0			1.661 342.80 0.0			1.661 340.50 9.0			1.661 340.50 121.10
2 2 2	18450.0	1.013 288.00 0.0	3 Q Q	18450.0	340.70	₹	18450.0	1.667 340.70 0.0	о О	18450.0	1.661 340.70 0.0	α Q	18450.0	1.661 339.60 0.0	3 Q	18450.0	1.661 730.60 118.50
MASS ELOW PATE	000	113 1.013 10 288.00	FLOW PATE	• 2000	.461 1.661 .70 339.70	NW RATE	000	61 1.661 0 228.70	OW BATE	000	61 1.661 0 338.70 0.0	AT A WC	000	61 1.661 0 338.70 0.0	NW BATE	.7000	7 1.661 0 338.70 0 116.10
MASS FL	41.7000	113 1.01? 10 289.00	אמלק בו	32.0	33.1	MASS FLOW	0002*28	6! !.66! 0 335.70 0.0	MUTE STOM	32.2000	61 1.441 0 335.70 0.0	MASS FI OW	32,7000	61 1.66° 0 337.00 0.0	WASS FLOW	32.7(51 1.667 0 337.00 1 113.30
TATION 12 Mach Number	0.5000	1.013 1.013 288.00 288.00 0.0 0.0	TATION 13 Mach Nimre	0.8000	1.661 1.661 334.80	TATION 14 Mach Nimrer	0.4000	1.661 1.66! 334.60 334.80 0.0 0.0	TATION 15 Mach Nimber	0.000	1.661 1.661 334.60 334.80 0.0 0.0	TATION 16 MACH NIMMER	0.5000	1.661 1.661 335.00 335.70 0.0 0.0	TATION 17 Mach Nimber	0.5000	1.661 1.661 335.00 335.70 120.30 115.80
TO AXIAL STATION Station Mach N	0.0410	1.013 288.00 0.0	TO AXIAL STATION 13 Station Mach Mima	0.0715	1 1.661	TO AXIAL STATION 14 STATION MACH NIJMR	0.000	1 1.661 334.60 0.0	AXTAL S	0.1150	1.661 334.60 0.0	AXIAL S ION	0.1350	1 1.661 334.80 33 0.0	AXTAL S	0.1550	1 1.661 334.80 33 126.00 12
TER STA	. .	3 1.013 284.00 0.0	INDUT TO	ċ	1.661	Thout TO	°	1 1.661 734.60 0.0	nt tuani Tep cten	•0	1 1.661 334.60 0.0	INDIST TO TIP STAT	°C	1 1.661 334.90 0.0	Thout Th TTP STAT	•	4 1.661 334.90 132.60
TIP RADIUS	0.2490	13 1.01. 288.00 0.0	ADTUS	0.2430	1 1.661	APTUS	0.2400	1 1.66 334.70 0.0	TIP RADIUS	0.2395	1 1.66) 334.70 0.0	S	0.2420	335.00 0.0	RADTUS	0.2440	335.00 117.70
d d i t	•	2 1.013 288.00 0.0	TIP PADIU	0	1 1.661	TIP RAPTU	0	1 1.661 334.70 0.0	a clt	0	1.661	TIP RAPIU	С	1.661 335.10 0.0	4 41+	o	1.649 335.10 1G7.20
HIB STATION	0.0200	13 1.013 3 288.00 0.0	STATION	0.0700	51 1.661 335.20	STATION	0.0960	1 1.661 335.20 0.0	STATION	0.1150	1 1.661 1 335.20 0.0	STATION	0.1350	1 1.561 335.30 0.0	STATION	0.1550	8 1.645 335:30 91.90
8		3 1.013 288.00 0.0	HUS		1 1.661	HC/8 S		1 1.651 336.30 0.0	HUPS		1 1.661 336.30 0.0	HUR S		1 1.661 335.50 0.0	HUR S		1.638 335.50 76.30
RADIUS	0.1100	1.01 288.00 0.0	RADIUS	0.1340	334.10	RADIUS	0.1420	2 1.661 338.10 0.0	PADTUS	0.1450	338.10 0.0	HUR RADIUS	0.1470	235.70 0.0	HUR RADIUS	06110	335.70 60.00
HUB	8 –8	TOT PRESS TOT TEMP	HUR	İ	TOT DRESS	HJA		TOT PRESS	H138		TOT PRESS TOT TEMP	RUH		TOT PRESS TOT TEMP	HUA		TOT DRESS TOT TEMP

INDIT TO AXIAL STATION 19

	1.661 342.40 126.10		1.661 342.40 125.20	1 1		342.40 62.60			1.620 347.40 0.0	T I		1.620 342.40 0.0			1.620 347.40 0.0	000
	1.661 341.40 123.00		141.40 122.10			1.638 341.40 53.00		i	1.620 341.40 0.0		!	1.620 341.40 0.0			1.620 341.40 0.0	0.050 1
1	340.50 120.20		115.30			1.634 340.50 39.80			340.50			1.620 340.50 0.0			1.620	000.0
RPM 18450a	339.60 117.50	18450.0	14656 339.60 101.30	Σ. Q.; Ω.	18450.0	1.629 339.60 27.30	X	18450.0	1.670 339.60 0.0	3	18450.0	1.620 339.60 0.0	Ø 8	18450.0	1.620	0.840
RATE	338.70 115.20		12652 138.70 87.80	P A 1	•	1.625 228.70 15.00	BATE		1.620 738.70 0.0	α 4 τ π		1.620 338.70 0.0	0 A + n		1.420 328.70 0.0	n.800
32	337.00 112.50	32.7000	1.643 337.00 67.40	ASS FLOW	32.7000	1.620 337.00 0.0	MASS FLOW	32.7000	1.620 337.00 0.0	SS FLOW	32.7000	1.620 337.00 0.9	ACS FLOW	32.7000	1.620 337.50 0.0	002*0 00
	14.80	66	1.635 335.70 47.50	7	c	1.620 335.70 0.0	g:	0	1.620 335.70 0.0	M	0	1.620 335.70 0.0	2	_	1.620 335.70 0.0	500 0.600
, ACH	335.00 3 99.60 1 TATION 19	0.5000	1.627 335.00 21.50	TATION 20 Mach niimber	0.5000	1.620 335.00 0.0	TATION 21 Mach Numbe	0.5000	1.620 335.00 0.0	STATION 22 MACH NUMBER	0.5000	1.620 335.00 0.0	TATION 23 Mach Niimred	0.5001	1.6?0 335.00 0.0	.400 0.
	334.80 79.50 IXIAL S	-	1.620 334.80 0.0	AL S	.50	1.670 734.80 0.0	اد د	400	1.620 374.80 0.0	AXIAL STA TION MA	00	1.620 334.80 0.0	S	00	1.620 334.80 0.0	0.300 0.
0.1750	334.90 33 54.90 7 INPUT TO AXI	0.1	1.620 334.90 0.0	NOUT TO AXI.	0.21	1.620	NPUT TO AXIA	0.2	1.620 334.90 0.0	INDUT TO A TIP STATI	0.2600	1.620 334.90 0.0	NPIIT TO AXIAL TIP STATION	0.2900	1.620 734.90 0.0	0.200
_	0 335,00 0 23.60 TABTUS	2480	1.620 335.00 0.0	S	2480	1.620 335.00 0.0	ν .	2480	1.620 335.00 0.0		4 A O	1.620 335.00 0.0	H	480	1.620 335.00 0.0	0.150
TJP RAF 0.2	335.10 8.00 TIP RAE	0.2	1.620 335.10 0.0	TIP RADIU	0.2	1.620 335.10 0.0	TIP PADIU	0.2	1.620 335.10 0.0	TIP RADIUS	0.24	1.620 335.10 0.0	TIP RADIUS	0.2	1.620 335.10 0.0	0.100
STATICN 0.1750 20 1.620	0 325,30 0.0 0.0	.1950	1.620 335.30 0.0	STATION	2150	335.30 0.0	STATION	2400	1.620	STATION	0.2600	1.620 335.30 0.0	STATION	.2900	1.620 335.30 0.0	0.050
HIJR ST/L	335.50 0.0 HUR STA		1.620 335.50 0.0	HUR STA	c	1.620 335.50 0.0	HIJB STA	0	1.620 335.50 0.0	HUR STA	•0	1.620 335.50 0.0	HIB STA	•0	1.620 335.50 0.0	0.0 NO.
R AD TUS 0.1500	335.70 0.0 0.0	.1500	135.70 0.0	RADIUS	.1500	1.620 735.70 0.0	RADTUS	*1500	1.620 335.70 0.0	RADIUS	0.1500	1,620 335.70 0.0	3 AD TUS	.1500	1.620 335.70 0.0	DEF INIT
HUB PRESS	I TEMP		T PRESS	K K	0	PRESS	HUB 4	00	PRESS	HUR R	0	PRESS	HUB	Ó	PRESSTEMP	STREAMLINE DEFINITION
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VR 0.0 1.3662596	2.1015692	2.4477154	2,7364960	1.9734584	4789704	1.0799446	0.6441677	0.3368263	0.1641465	c • c	7 > 2	0.4278703	0.4278268	0.4277867	1/4///4.0	86177740	4 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	0.4276075	0.4777086	0.4277142	0.4277253	0.6277253	016/1/310	01677240		OHO	1.707518	1.45021.1	1306000		1.1204348	1.1204367	1.1204310	. 1204	1.1204243	706	1.1204105	.120416	20416	1.1204166
VM 142.08244 142.07168	142,95544	142.93491	142,02305	142.92668	142.03030	142.93205	145,03567	107 60 671	142.03744	2.0374	ž	0.51627	0.60848	0.68836	0.55.7.0	46760	0.447	1.14140	1.22895	1.31050	1.38742	1.47478	1000 ·	22Ch4-1		RANC	1000000,56250	0.547541000000.56250	0.25.000001.5/47.0	7. 44777 988000 44250 1.02533100000 44250	1.005071000000.56250	0. R96671000000.56750	0.741041000000.56250	0000,56250	32911000000.56250	0.280261000000.56250	0.204591000000.56250	0.124211000000.54250	3	000000.56250
V 142.08244 142.07168	142.05544	142,02201	142.02445	142,02669	02020-671	162.03205	147.03567	142 03744	147.03744	147.03744	X > 3	0.42787	0.427R2	0.42774	0.6776X	74774 O	C 4 4 C 4 C 4 C 4 C 4 C 4 C 4 C 4 C 4 C	0.47764	0.47769	0.42770	0.42772	0.4777	,	- 6		٠.	0.0	0.54754100	001,5744.0	1.02533100	1.00507100	0.89662100	0.75105100	0.592841000000.5625	0.43291100	0.78026100	0.20459100	0.12421100	06 F R 01	0.0
BETA 9AR 124.04436 135.32161	141.57729	148.78342	153,01431	157,00532	159,63330	160.05241	165,04387	14.204698		163.76000	> 5	4.	•	0.42779	0.42775	0.42772	70//400	0.42770	0.42771	14460	4.	0.42773	•	•	**************************************		0.0	4.00804	\ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \	F.00022	10,00085	10,00097	10.00064	10,000?2	•	0,000.6	4.00005	•		4.00001
BETA 124.04434 135.32292	141.58029	148.78746	153.01785	157,09701	150,63479	160.05200	167.04410	05070*741	1 6	163.76000	3	-96.60394	-144.58490	-190.71829	-200° 88014	725, 45,805	17x0.57.744	-352-67310	-385,01343	-412,99755	0	453.0020	-466.56616	P0004.874-	1410.1414		288.00000	288,00000	00000	288,00000	288,00000	288,00000	288.00000	288.00000	284.00000	288.00000	288.00000	α.	288.00000	288.00000
A TCTAL 340.23535 340.23535	240 -23535	240.23535 240.23535	340.23535	340.23535	340.21515	340.23525	340.23535	140.7.044	340-23535	340.23535	3	172,5581!	203.33710	230.02235	253.04121	275, 78900	414°01364	381.46118	410.68774	417,04770	463.64231	475.06240	487.97021		711.1403H	-		1.01350	01.250	0.5.10.1	1.01350	0	6	.01	1.01750	1.01350	1.01350	1.01350	~	5
ALPHA 89, 90097	199.0007	70000 08 80 00007	90,00007 90,00007	49.00097	80°0000	80.0007	40°00097	89° 99° 97	80,00007	80,00007	5	0.0	0.0	0.0	0•0	0.0	0.0	- c	c • 0	0.0	0.0	0.0	0.0	0.0	0.0	STAT TEMP	277.82690	277.82837	277,83081	277.83232	277.83406	277. 83521	277.83472	277.83423	277. 8339R	277.83350	277.83350	277. 83325	4322	277,83325
ALPHA BAR RG. CGOST 89.00007	89,00097	70000 68 89.00007	80.0004 80.0004	80,00007	80,00007	U	80,00007	0 (20000.00	•	×>	142.08244	142,94515	142.94006	145,021 AC	142,01103	142.90794	147,40175	3	42	147,01796	145.03474	142.02704	~ 1	42.03744	STAT PRESS	0.89364	0-89766	0.89368	0.80370	0.89373	0.8933	0.89372	0.89372	0.89372	N. 80171	0.89371	0.89371	0.89371	0.89371
A STATIC 334.17236 334.17334	334.17456	334,17578	334.17725	334-17700	334,17676	334,17651	334.17627	334.17627	534-17603	134.17602	NEGY	•	12,17362	21,21402	28,74154	35,33066	46.70258	56.49356	73-17368	80.52478	87.39366	90.67471	93.86475	£ 0026 95	100,00000	=	96.60354	144.58589	180.21828	20d.RACI4	280 67066.	319,26578	353,67310	385.01347	413,99755	441.06079	453,99202	466.56616	478.80908	490.747RD
RADIUS 0.05000	0.09328	0.10863	0.14527	0.18305	0.19027	0.21427	0.22828	80462	0.24148	0.25400	31118	0-0	0.02483	0.0432R	0.05863	0.07207	0.00527	0.11525	0.14927	7	0.17828	0.18408		1078	0.20400	X-VALUE	-0.26000	-0.76000	-0.26000	-0.26000	06067-0-	-0.26000	-0.26000	-0.26000	-0.26000	-0.26000	-0.26000	-0.26000	-0.26030	-0.26000

٥>	O. 2PRI 502	ď	2. FROURRR	2.01A102R	3.0747810	2.0110422	7.5415478	2,1021691	1 4441340		******	0.7740275	0.5690317	0.4714807	0.1824148	0.0000369		F/AE	0.4200766	0.4194647	Inchi/4.0	0.4747484	0.4244992	0.4268606	0.4283214	0.4203166	94000640	0.4304646	0.4307469	0.4308254	0.4704962	0,4300240	0.4300351	0	1 1 2 2 2 2 0 0	1,1242112		1.1224804	1,1219147	1.1208191	1.1291429	1,1106823	1.1103628	1,1101502	1,1100186	1,1190775	1180680	1,1180265	
3>	140,46753	140.26930	140.01307	141.40156	141.05976	142,65601	143,12785	143.44855	143 47024	7.43 P10.7		144,90947	141.02795	143.05757	143.0462R	. 0400	3	R (4,000.0	0.60460	* / CLL* C	445 4 1 6 1	0.42623	0.04482	1.04952	1.14474	1.23164	1, 21,200	1. 38048	1.42615	1.46799		1. 53083	2049	-2.7266E	4, 21, 21, 2	3.56026	2,79605	4.23400	5.41802	7.08657	0.35758	12.7487]	19.34706	20,00050	41.56606	45.04783	142,17205	-4515,10022
>	140.46753	140.26039	140.01207	141.49156	141,05076	147,65601	142,12785	143,44855	142,47024	2918 52		40000	4	142,06747	. 9663	143.06000	2		0.4700	•			0.47460	0.42477	0.42825	0.42027	0.420ng	0.43045	.+	•	0.43089	0.42002	0.43004	Ü	25833	0.74008	1,00245	1,22144	07076.	1,16025	1.01747	0.83967	0.45440	9.4775A	719020	0.22651	0.14703	0.07240	
RETA BAR	124.51750	125,07763	147,10805	146,14052	149.04783	153,14134	155,01402	157,94548	159,56577	160.86150		75 (145 - 191	2	162.85721	144.26741	162,64995	3		•	•	30000	0.000	0.4.4.0	0.42684	0.42832	0.42032	000170	0.43046	4	•	4.	0.43002	0.43004	A ADEA		5.04020	5.06340	5.04705	F.02163	10.03608	10.00429	966800	0.94714	0.05670	940560	4.97331	4.072KK	4.07275	4.9722A
RETA	\sim	F. 9799	42,1140	146,14653	140.07373	153,14616	155.01740	157,94767	159.56703	60.8622	161 02.020	7	162.41653	167.95727	٠,	163.64995	Š	70507 70-		٠,	210 00354	05-00-017	/ \$C\$ x \$ \$ \$ \$ \$ \$ \$	DC4P4.185-	-220-17700	112670932	-385. K1228	-414.42529	-441.34424	-454,20093	-466.70100	-479.97573	-400,74780	TOT TEMB	288,00000	288,00000	284.00000	288,00000	788.00000	Œ	288,00000	α	288.00000	ď	2 8 8 . 00 00 0	288.00000	α	288.0000	288,00000
A TOTAL	340.23535	240.23535	340.23535	340.23535	340.23535	340.23535	340.23535	340.23535	340.23535	740.23535	260 27525	0.0000000000000000000000000000000000000	540 • 2535	2363	.2353	340.23525	,	170 69006		. 0		776 1 1700			1011/046			28.67000		46240	00002.88	00.04834	511.42003	TOT DOFSE	1.01350	10.	1.01350	1.01350	1.01350	1.01350	1.01350	1.01350	1.01350	.01	1.01350	1.01350	1.01350	1.01350	1.01350
Al. PHA	80,00007	80. 90ca7	80,00001	49,00007	89,00007	89, 99997	80. 90c97	89,90007	40,00007	80,09097	70000	00000	10000000	4 0 0 0 0 0 M	80.0000	R9.09097	5				0.0		•	•	C 0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	STAT TEMP	278,18164	278,2002ª	278,11014	278,03784	277,07192	277. R7305	277,80615	277,76050	277.72876	277.70752	277.6945R	277.69043	277-69774	277.6P62R	277.68570
ALDHS 940	0000.0	3300.0	0.000.0	2200.0	80,00007	ď	c.	Be cooct	ď	ď	, 0		•	,	80.00007	Ac. 00007	>	40.4	7576	40.042	141.45041	141 62449	7477-141	000000000000000000000000000000000000000	143.101.4	*******	4.3.6.60 R	42.82.54	2400°24	44°0347	43.0570	43.6667	4	T PBESC	4.	5	C. A9693	~	-	۷.	_	ç	4	2	ur.	0	<u>_</u>	ş	Ľ.
ST	34.	34.	34.	34.	334,25552	34.	34.	34.	34.	34.	4	4		•	4	7.	NAGA	0-0	12,31602	21.43272	78.99681	25,5663	70670 77	10070707	16671.00	77 77 77 77	13.32643	80.63669	87.46606	727	3. 499	180.40	1 00.0000	1	46609.90	145.13982	181.07048	210.88356	236.89542	281.65629	320.17700	354.43311	385.61328	414.42529	4	S	9	478.87573	490.74780
P An TUS	.0533	.0751	.0037	1001	0.12762	1458	1657	.1834	1 oc E	.2145	2296	2250			7478	, 2540	S-VALUE	0	251	427	0501	0.726		746	712710	# C 7 P	. D 4 I	1645	• 17R4	1850	1915	1 0 7 R	.2040	- X-VALUE	_	-0.23300	-0.23030	-0.23000	-0.23000	-0.23000	-0.23000	-0° 2 3000	-0.23000	-0.23000	0	-0.23000	-0.23000	-0.52000	-0.23000

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9>	07470	3,0253893	4.6306201	4.7423630	4.5804210	4.0854712	3.4520950	7,04001,7	2,1424656	1.5593096	0.9960791	0.7200188	0.4746554	0.2311084	-0.0001-02	*	0,0000	40710	0.4142221	6199343	422250	424022	2030000	0.4217073		0 4 4 4 5 C C C C C C C C C C C C C C C C	10-04-40-0	0.4344RF7	7840464	34 7 84	340	0.4348783	CHa	1.1225783	1.1207369	1,1245421	1.1244516	1.1220482	1.1207005	1.1104705	1.1195303	1.1179237	1.1175251	1.1172733	1,1171042	1.1171350	1.1171017	1.1170012	
5	124.25573	136.30038	128,60764	140.09854	141,14043	42.67670		44.24768						144.21242	2205	3		ָ מיל	α	7	Č	97876	99690	1,007	7.50.0	4-5-7-6	5	0695	1.47 H45	• 44 50	Œ	1.51240	P A N C	Ľ.	1,02570	1. 12324	1.62096	1.95774	2.60400	7.64576	0 2 0 5 0 5	4.87044	2	15,53547	5	31.27348	41.1514	, ה ה	
>	134,2553	3002	1 28 60764	140.0984	141.16043	142.67670	143, 42278	164.24769	144.66667	144,04107	145,11414	145,14801	144,20054	24	4,720	XAM	10000	0.4040	0.41400		4220	0 42475	72024 0	100000	1000 C	•	7	4364	4 146	1767	۵ ا	0.44AA	202	-0.5577	1.45012	1.01451	1.03084	1.85024	1.64096	1.37729	1,11212	0.95664	0.61402	0. 402 20	ď	1872	1100.	-0.0006	
AETA BAR	125,72705	137,02514	142.79575	146,60147	140.37512	153,25456	155,020RF	157.01008	150.47316	160,74701	161.81274	162.78490	162,72200	163.13135	3.513	3	•		0 61622	0.41802	0 42225	7 7 7 603	•	•	•	016550	٠,	0 4 3 4 4 0	•	0.43478	. 424B	0.47488	F AGEA	0.0	5,71525	5.15604	4.10317	F.06242	10.06063	72600°0	0.04852	0.0102	0.89970	9,44755	.9402	389	380	4.02770	
ATUR	125,73836	137,03702	142,91116	6.6165	140.38835	153,26402	155,02703	157,01385	150.47523	160.74800	161,91313	162,28511	162.72206	163,13125	162,51357	i		146 20203	102 601	-212 48257	1330 44051	1.001.101.1		C + C + V U C	7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7	-286.37695	-414.0777R	_	-454.46143	-466.97183	-479.05874	-400,74780	TOT TEMP	288.00000	289,00000	284.00000	788.00000	288.00000	288.0000A	288,00000	288.00000	289,00000	c	28	288	28	Œ	α	
14774	7357	340.03536	240,23535	340,23535	340.23535	340,23535	340.23535		340.23525		275	7353	340, 23535	340, 23535	40	3		100 000 0000	770 7770	356 51316	30000 660	20070	10010 011	1,020°74		4:2,57178	9	464.00554	77.0A7	9226	00.4	1898 9	THE DRESS	4F 10.	1.01350	.013	1.01350	~	1.01350	.012	.013	c	1.01350	1.01350	.01	1.01350	.013		
4	40,0000	89,00007	99,09c97	7999998	89,00007	80.99997	80.00997	89°ccc97	199 carg7	49,00007	49, 90007	89, caca 7	89,00007	76000 68	89.90cG7	5		000	• •	•	•	•	0.0	c. 6	0.0	0.0	0.0	0.0	0.0	o.0	0.0	υ • υ	STAT TEMP	279.03076	278-75537	278.43094	279,23315	278,08225	277,87036	277, 73535	277,64600	277,59569	277.54614	277.57124	277, 51343	277,50757	277.50415	277-50317	
	PO COCO DO	70000	40,00007	80,00007	16000 08	89,00007	800000	90 00c01	80,00007	80,00007	90 00001	80,00007	89,000,7	80,000Q7	RC. GOCG 7	3	Y	134.24896		30010 071		11 ch 0 * T * T	01614.741	143.58270	144.77050	144.45050	144.92265	145.11072	145.16707	145.20879	145.23224	145,22052	STAT PDESS	10,40727	0.90414		0. PGA77	0. RO6 F 3	0. RO413	0.89261	0.89160	0.80793	0.9004R	0.89020	0.49011	50008.0	10066.0	0-89000	
	226 OCES1	236.73022	234,54077	334-41650	334,32642	234,19840	234,11719	224.06348	234.02724	334,00342	323,08828	227.06266	223,98022	777,07867	333.97754		10 N	0.0	10000	67010.17	01100	37.44.68	14626.14	57.03412	65.66745	73. 52129	80.77722	87,55623	90°19408	93.94261	00000*20	100.00000	Ξ	96.60354	146.29302	182,58052	212,48257	238,46851	283,1215R	221.38892	355 41910	386.37605	414.97778	441.69873	454.46143	466.87183	478.95874	490. 147RD	·
	PADIUS 0 OFOO	0.00000	1 400	0000	0.1244	0.14654	0.16535	7.18306	20001	0.21479	0 22461	0.22.0	77177	0.24790	0.25400		S-VALUE	0.0	57 C 20 0 0	0.0440.0	0.00	0.07344	0.00654	0.11635	9626 1.0	0 . 1 400 A	0.16470	0.17861	0.19522	0.19154	0.1970	0.20400	X-VALUE	-0.20000	-0.20000	-0.0000	-0.2000		-0.2000	-0.2000	-0.2000	-0.2000	-0-2000	-0-20000	-0.2000	-0.20000	-0.2000	-0.2000	
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SAWA D	FTAD	PP/(01-0)	OP/(oto-p)	10/00	T(pa/pa)	10/10	1101/01
1.40002	0.0	0.08310	0.05526	1.00000	1.00000	1.00000	1.00000
1.40002	0.0	0.05253	0.02468	1.00000	1.00000	1.00000	1.00000
1.40002	0.0	0.03111	0.01092	1.00000	1.00000	1.00000	1.00000
1.40002	0.0	0.01875	0.0052A	1.00000	1.00000	1.00000	1.00000
1.40002	0.0	0.01062	0.00248	1.00000	1,00000	1.00000	1.00000
1.40002	0.0	-0.00026	-0,00005	1.00000	00000	1.00000	. 00000
1.40002	0.0	-0.00663	-0.000RB	1.10000	1.00000	1.00000	1.00000
1.40002	0.0	-0.01067	-0.00115	1.00000	1.00000	1.00000	1.00000
1.47002	0.0	-0.01320	-0.0011P	1.00000	1.00000	1.00000	1.00000
1.40002	0.0	-0.01496	-0.00112	1.00000	1.00000	1.00000	1.00000
1.40002	0.0	-0.01634	-0.00102	1.00000	1.00000	00000	1.00000
1.40002	0.0	-0.01679	-0.00047	1.00000	1.00000	1.00000	1.00000
1.40002	0.0	-0.01467	-0.00092	1.00000	1.00000	1.00000	1,00000
1.40002	0.0	-0.01695	-0.000R6	1.00000	1.00000	1.00000	1.00000
1.40002	0.0	-0.01690	-0.00081	1.00000	1.00000	1.00000	1.00000

VP 6.4143686 9.323138?	8.40000AQ	A.0257380	6.0800181	4.941907	2.0044479	2.0670165	2.1200199	1.3524141	9028200	0.649053B	41434	0.0004891		0.3408170	0.3971106	0.40AQQ45	0.4140792	9.471070A	45018670	0.4276076	0.4255201	0.4375016			0.4308657			0.4401312	CHa	.157345	-	.128073	1.1257477	1.1224732	1.1202030	1.1181545	1.1147860	1.1158600	Ξ.	٦.	1.1147470	1.114669R	1.1146788	1.1146,21	
VM 114.62479 173.02946	134.85400	130,17584	143.08453	144.50702	145.44970	146.09347	144.50514	144.74339	146. 94354	144.80577	146.92363	144,92985	3	.4444	. 5062	•	•	•	•	•	1.15357	•	1.37016	•	•	•	1.50103	1.52453	2 A D C	0.27141	.6330	0.86113	.0803	. 31 51	. R 207	2.46445	3-30556	4.57531	4427	10.75006	15.21457	758	2	-377,01597	
V 114.63420 120,6346	124.85400	α.	142,08653	44.5070	604	46.04	46.	2	7	0	756	* 46. 97AA5	ЖЛЖ	9.74028	F. 306.0	0.40A0A	0.4149	0.42051	0.47781	95CE7*U	75 75 7 O	0.4374	140F7.0	0.42060	0.430RK	•	0.44011	0.44013	202	3.2076K	4.01A77	3.44065	3,30564	7.0000	120270	1.05077	u.	1.16613	n. 87012	Ľ	7.4	1226	1261.	0.00010	
8FT& BAR 130-23404 138-23570	142,54477	147.08720	144.68900	155,89932	٠,	150.33835	160.58588	1.63	162.10599	•		•	> 1	0.34082	0.30711	0.40891	0.41608	0.42108	0.42819	0.43260	0.43553	0.43750	0.47881	4.	4.		0.44011	0.44013	A TO A	0.0	5.52822	5.2400°	5.150RF	5.04210	5	14640.0	0.40700	F 2 5 3 4 3	0.87457	C-80707	4.8984D	ဗိ	895	4.89523	
RETA 130.27835 138.30585	143,60007	147.13084	140.72281	155,01170	157. 92533	150,34222	60.5A77	161.62838	62.10		2.94	7.3323	3	96000.40-	-149.07260	-145,25151	-215.0287P	-240.82965	-285.12769	-323.03031	-356.73525	-347.24281	-415.69702	-442.15674	-454.79639	-467.0A9R4	-479,06494	-490.74780	TOT TEND	288,00000	α	α α	88.	ρ. Θ.	ά	α	Œ		ďα	ά. α.	ď	α	248.00000	α	
A TATAL 340.23525 340.34535	3635 025	340.23535	360 33535	340.23525	340.23535	340,02535	340,73635	340.23535	340,23535	340.23535			3	_	_	~	256.13916			367.pa794					477,91504	480.64404	501.CPR62	512,27075	SYMME TOT	.013	.01	.0	1.01250	1.01350	1.01350	1.01350	1.01350	1.01350	1.01359	1.01350	7	1.01350	1.01350	•	
ALPHA 89, 90007	A9.00097	80,00097	80,00007	7000 00	90 acac7	80,00007	89,00007	PQ. C0097	40,00007	80° ccc 07	70000 ng	80.90c97	9	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	STAT TEMP		279,19385	278,68019	778.24122	27R. 12622	277. A1203	277.60889	277,47290	277. 3ADA6	277,31934	277.28174	277.27002	277.26245	277,25830	277.25757	
ALPHA RAP RO,00007	80°00007	ac.acqc1	9 6	100000000	000	89,00007		2000	O	0000	pc occor	An. 00007	× >	-	7073	~	138.c4424	140.60063	142,05688	- 4	145 AG6P7	146.05331	146.48087	146.75714	146.84018	•	46.0237	6.0289	STAT DEFSS	7.0257	0.01012	.003	PC Q A	1700.	PO34	. 8011	APAR.	٠.	٠.	0. 44752	۳.	۳.	C. AP 725	. P.R.7.	
A STATIC 236,35059	434.68506	334.49365	724.35875	334.16333 334.04103	774,04107	333,90381	433, 86694	333,84424	333, 83716	222,83252	333, 83008	05028°666	NDON) P 4	717	3.2	367	. 7	2	404	8	241P	87.66CP3	3705	922	7.0332	100.00000	=	45,000.46	149,97260	185,25151	215,02878	240.82C65	285.12769	223,03521	356. 73525	387.28291	415,65702	442.15674	454. 79639	467.0 ACP4	79.0640	90.7478	
PANTHS 0.05020	0.37710	0.11131	0.12466	0.14/20	12791.0	0.20051	0.21516	0.22885	02220	7.74176	0.24705	0.25400	211 W 7- 3		•	0.04570	0.04111	0.07446	0.797.0	0.11701	7 3445	15021	0.16606	3-1 7865	0.18619	0.10156	0.19775	٠.	X-VALUE	-0-17000	,		-0.17000	-0.17000	·	_	_		-0.17000	-0.17000		-0.17000	-0.17000	-0.17000	

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VP 20.0431696 16.3023682 14.2274617 12.5408897 11.2101583 8.0031059	5.5456047 4.2074280 2.0811096 1.87828492 1.3787286 0.8851172 0.429357	MVM 0.3405317 0.4056317 0.405635 0.405606 0.4414617 0.4471815	0.4508027 0.453847 0.4559467 0.4559467 0.4555048 0.455560	1.156648 1.1356648 1.1356662 1.1257987 1.1214014 1.1102603 1.1076411 1.1077861 1.1077861 1.106783
VM 115.23050 134.13176 130.14610 142.18503 144.31890 147.35640	150,38264 151,15821 151,67928 152,00102 152,10643 152,2841	MW 0.46721 0.46721 0.70426 0.7820 0.86034 0.96959	1.16587 1.25055 1.40426 1.43994 1.643994 1.50833	0.25519 0.25519 0.50501 0.67333 0.67333 2.01421 2.77115 8.67364 11.57164 11.566030
V 115.72050 134.13176 136.14610 142.18503 144.31890 147.35640	150,38764 151,16871 151,67028 157,00102 157,17003 157,2861	MVX 0.3460 0.04124 0.04134 0.6564 0.6664	0.45058 0.45217 0.455217 0.45501 0.45770 0.65770	0.40198 6.00198 6.00198 7.00198 7.00198 7.1086 7
BFTA BAP 122.15P40 130.063&4 147.06417 140.46158 152.91524	157.24905 158.75227 158.90277 161.05333 161.65683 162.37680 162.37690	4.0 0.42067 0.470651 0.47590 0.47590 0.43207 0.44144	0.45080 0.45080 0.45506 0.45505 0.45503 0.45650	# Apra Apra Apra Apra Apra Apra Apra Apra
RETA 132.43042 130.7437 143.04670 147.14690 140.43735 152.05851	157.26215 158.76275 159.00127 161.05470 161.96554 161.96524 167.37686	WIJ -104.44658 -154.44658 -100.12096 -210.31737 -244.62959 -278.14868 -325.42578	-358.58780 -388.77441 -416.68384 -445.78347 -457.38867 -467.31143	288,00000 288,00000 288,00000 288,00000 288,00000 288,00000 288,00000 288,00000 288,00000 288,00000 288,00000
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57471C 236,31006 234,90552 234,24063 334,05737 233,79199	23 23 23 23 23 23 23 23 23 23 23 23 23 2	9 SDAN 0.00 13.02875 22.21132 25.316439 47.57652 57.22182	65.80258 72.61345 80.83517 87.58826 90.81577 93.95547 97.01474	104.33228 154.64652 190.12964 219.31737 2246.4250 288.14868 325.42578 358.58789 358.58789 416.68384 442.78247 447.21143
2401US 0.05600 0.08662 0.0862 0.12662 0.1663	0.18F61 0.20123 0.22clp 0.23c63 0.24141 0.24141	S-VALUE 0.0 0.0260A 0.04442 0.07563 0.00513	0.13161 0.14724 0.16167 0.17518 0.18763 0.18763	x - VA 14 15 15 15 15 15 15 15

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¥ >	124.60940	32.4705	137.00211	141.20772	143,55585	146,01670	148.07000	150, 21824	151 2221	11777	0/4/8-761	152.17820	152, 28340	152.34691	٠,	152, 36200	3		0.6279	0.71057	0.70577			1.08147	1.17262	1.25618	1.22286	4049		1.47592	5090	1.54152	0	7000	76444	0.6225R	0.77810	0.94437	1.33233	1.81611	2.45782	3. 37651	4.01944	A . 30036	12,70243	25.00371	277,56763	144.10771
>	124. 60040	122.47052	137.00211	141,20772	142 55585	146.01470	148,97009	150.21874	151 22211	12 017 12		157.17830	157,78340	u .	152,37199	00695,520	× > 20	0 2576.6	7886	0.40744	0.41894	007640	0.43859	. 4	0.45016	0.45222	0.45522	0.45444	0.45680	0.45701	, 4	٠.	C L	1 K 822 R3	10.74874	8,66043	7,785,20	6-76152	4.76211	2.64764	2.77769	2,05133	1.42078	700066	0.66815	0.43832	A777C.0	0.00266
RETA RAP	133.39412	140.07743	145,05702	147,94281	150.13220	153,31348	155.61578	157.39694	158.83685	160 02025		61490.191	76166 (4)	-	2.3676	162,75203	>	~		0.41214	0.42237	0.47965	0.4400	0,44649	0.45049	0.45351	0.45537	0.48450	্ব	0.45703	4		A00.4	c	5.65691	5,30269	5,23952	F.1.3493	1082	0.04803	0.95136	O. TARK4	0.74784	cr£c1.0	4.45446	4.85730	4.85122	4.85119
RETA	134.48070	141.47240	145.36488	144,15129	150.28011	153,30250	155,65045	157,42072	159,84020	160 04405	CO110 (0)	101.00747	161.52434		162.36774	162,75203	3	-117 95683	-162.45581	-197,34252	-225.47982	-240.07734	-297.28747	-328,643Ag	-261.06226	-300,41014	-417,98145	-442.59692	-455,84700	-467.77246	-470, 19771	-490.74789	TOT TEMP	288,00000	2 RR 20000	788,00000	288.00000	284.00000	288.0000A	288,00000	288,00000	288,00000	28° 10000	Œ	284.00000	α	288.00000	288,00000
A TOTAL	340 . 23535	340.23535	340,23525	340,23535	340, 23525	340. 23535	340, 23535	340.23525	340,23535	360 235	36366 076	140.140.1	C6C6 - 04.	240.2454	40.0352	140,01535	3	171,58128	210.30542	240.76743	266.04545	780.74538	327.12915				444,6940	468, 07363	90.A1	401 05561	502,02003	~	TOT DECK	1.01250	1.01350	1.01350	1.01350	1.01350	1.01350	1.01350	1.01350	1.01350	1.01350	1.01350		1.01350	1.01350	1,01350
AL PHA	4000000	80° 00001	80.00007	89.00C07	80°09c97	80,00097	80.00097	90,00cg7	40,000,08	40.0004		10000	10000 ok	Locato tox	80.000	40° 00001	ΙΝ	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0•0	STAT TEMP	290,26221	279.2675A	278.53687	278,07788	277,74512	277.25078	276.05703	276. 75625	276.62061	276.53101	216.47632	276.46021	276.45068	276.44678	276.44849
b WHal	8c. acca 7	5000	ge cocc 7	5000	Po.ccoc7	9000	4c, 00007	40.0ccc7	40,000.07	΄ υ	700000	, (•	•	٠,	80,00007	×>	110.07664	30.3462	36.3207	40	42. EC	4	æ	50.142	141.12520	151.76977	50	121	157,34245	152,37789	50°	v	AE110.	96060	00166	PCK47	0272	727	8388	R164	201 ₂	7914	7952	7835	A24	0682	0.87822
STAT				334,32324			333.64PER		223.44604			1	r, (<u>.</u>	733,34131	Š.	A SPAN	0.0	12.23686	21.33118	28.97123	25.44170	46.78534	56.53502	65.22737	73.15240	Ö	87,35725		93° 83046	96.95	9	5	117.85683	163,45581	197.36252	225.47882	240.01724	292,28247	328.64380	361.06226	340.61914	417.98145	443.5662	455.847c0	467.7724K	470,39771	400,74780
An IUS	.0610	.0P46	1023	. 1167	, 1204	i si i	0.17013	1860	2021	2163	2006			747	1 7 7 7 °	2547	S-VALUF	0	0.02362	0.04117	0.05472	0.06840	0.040.0	0.10011	0.12FPn	0.14118	0.16574	0.16860	0.17454	0.14111	J.18713	0.10300			00011.	.11000	.11000	1220	370	0.11000	1000		1300	1220	1000	11000	0.11000	

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10-010)/00	-0.0AR72	0.00964	0.00 50	0.00340	0.0022R	0.00098	0.00039	000000	-0.00006	-0.00014	-0.00015	-0.00014	-0.00012	-0.00010	-0.0000a
19-19/40	-0.16534	0.02367	0.01706	0.01312	0.01007	0.00568	0.002R4	0.00091	-0.00067	-0.00172	-0.00222	-0.09221	-0.0000-	-0.00170	-0.00142
ETAP	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
SAWA	1.47002	1.43002	1.40002	1.40002	1.40002	1.40002	1.40002	1.40002	1.40002	1.40002	1.40002	1.40002	1.40002	1,40002	1.40002

g >	43.4096619	23.6834106	27. 4972882	23.8422394	20.6958773	15.8085357	12,2691612	0.3464365	4,0097004	4. R20R044	2.001032	incompt of	62/4947 • V	1.3730574	0.4420782	-0.0261401	¥ > ¥	19173020	1 0	0.426494	1455767	0.4434553	0.6539018	0500000	0.460544	C	0.4667410	1 () () () () () ()	0.4600A07	0.4693007	N. 4606FR7	0.4609072	0.4701356	2	1.1340230	1.1273400	1,120003	1.1164608	1.1120476	1.1080437	1.1040442	1,1020002	1,1017790		1,1006193	1.1004658		100710		T (1) (1) (1) •
3 >	132,47841	138.02740	142,52075	145.67274	147, 99673	151.34656	153,38936	5.4	155,45120	155,92160	156.20059	•			. 45 A	166.53415	3	0.54.704	0.64980	0,75383	0.82572	0,48952	1,00184	1 0000	1 18782	7.010	7 7 7 7 7	07446.1	•	1.44037	1.48247	1.51522	1.54708	PADC	0.60553	0.58773	0.60530	0.82277	O OPORR	1.30652	1,90000	2,03745	4.55057	7.36676	11, 11530	12,7200	13,20280	12,64084	11.52038	
	122.47841	128.02740	142,53075	145.67274	147,99673	151.34656	152, 18036	154.46483	154,451 20	155,02160	156.20059	2000	C#07 * 44 * 1	156. 3R213	C	154.63415	×>*	71217	40004	0.41822	7.054.0	0.43010	05 17 20	10037	100.240 0.4442	00777	X 244 C	70 64 4	0.46400	92047°U	0.46964	Œ	0.47014	n O	10,21094	14.12477	11,29522	0.41008	A 03840	6.02089	4.58707	3,46451	2.54760	1.77176	1.09721	75055	0.50307	0.22413	-0.000.1	•
LL.	135,87700	141,98268	145.54672	148.10968	150.09705	3	156.25674	156,99155	158.41568	159.61528	160,64291	710000	01:61.61	161.53759	9220° i	162,30892	>	149561	0.41253	0.47647	0.43622	0.44346	0.45390	0 66030	0.46428	0 66476	\$ 204.0	1,000	0.44.0	0.46940	C. 45075	•	0.47014	T AREA	0.0	5.63661	5.39707	5.2465R	CF851.3	10.10701	666300	0.94631	0.78660	0,75010	0,77900	4.85700	4.85536	4.85324	4,85144	•
PETA	137,53284	142,43200	146.06674	148,45753	150.34097	153.18758	155,32657	157,02021	158.43507	150.67474	160-66619	70701 171		42624.191	1.9337	162.30882	5	-136.50709	-176.55721	-707.7465R	-234-12131	-757 34390	-207,70736	-225 R2081	19816.248-	-302 03007	\$550 VEVI	70000T#1		-456.59644	-468.76196	-470.63B67	-490.747RD	TOT TEMB	244,00000	288,00000	288.00000	2 AP . 00000	288,00000	288.0000g	288.00000	288,00000	288,00000	288,00000	288,00000	288.0000	288,00000	288,00000	288,0000	•
A TOTAL	36362.045	340,23535	340,23535	340,23535	340.23535	340.23535	340.23535	36356.025	340.23535	340.23535	340.23535	36366 076	210 0000	440.005	. 352	76355.025	3	100.28802	24.10713	~		206 RK409	774,04977	144.47583	305, 60727	427 57170	200000000000000000000000000000000000000		40.40.40.40.40.40.40.40.40.40.40.40.40.4	417.06.744	403.68457	4.5110	515.1CA15	TOT DRESS	•	1.013 = 0	1.01350	1.01350	1.01250	1.01750	1.01350	1.01350	1.01250	1.01350	1.01350	1.01350	1.01350	1.01350	5	
ALPHA	40,00047	80,00007	Ag. cocca	89.00cg7	40,00007	49,000.07	go cocd7	49° C0007	79.00cg7	49,000,08	89,90097	10000	10000		49.00007	80,000,7	5	0,0	0	0.0	0.0	0.0	0.0				•	•	0.0	0.0	0.0	0.0	0.0	STAT TEMP	279,74660	278, 5197A	277.80111	277.44043	277,10083	276,60191	276.29199	276,09668	275,07534	275.00234	275, 85013	275 P4375	275,82081	275. P18R5	275_AC713	
ALPHA BAP	ŏ	80,00007	00000	_	80,000,0	8c. 99997	80,00007	80.00c07	Ag oded 7	80,00007	89,09097				5000	4c° coco 1	>	125,02495	122 85627	139, 77403	143 70837	44. 547	50.500	157.85780	154.28216	156 20765	7 7 7 L		7;	Ė	¥ 1	56.456	156.53415	STAT PRESS	0.0000	0.00146	96500	C. 88930	4	õ	0.87648	0.87431	0.87267	0.87216	0.87158	0.87151	0.P7137	0.87124	0.8711	
71.	3371		234.21054		333, 73535	733.43457	333,24805	333, 13013	232.05699	233,01764	122,96657	000000000000000000000000000000000000000	67116-765	332.46973	232.06265	235,05522			11,20087	20.05072	27.54742	34,10541	45.576.4	55.41707	F4.27817	77 30450	71010 07	- 1111	30 5 CO	40°47°	43.65149	96. RE349	100.0000	. -	136.597cA	176.55721	207.74658	234.12131	257.343cc	297, 79726	332.R3CP1	364,21851	392,93004	410,60352	444.61865	456.59644	468.26196	479.63867	490-74780	•
RANTUS	0.07070	0.00140	0.10754	0.12119	0.13322	3.15415	0.17229	0.18952	0.20238	0.21718	23013	10000	5 5 5 5 5 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6	0.24236	0.74P?F	0.25400	7-VAI 11F		0.02070	0.036 84	0.05040	0.06252	7,08245	97101.0	0.11782	3 1 2 2 4 0	30.20.00	C * C U * C	0.1545 0.170	194910	0.17166	0.17755	0.18330	X-VALUE	-0.0800-	-0.0800	-0.0800	-0.08000	-0.08000	-0.0R000	-0.0P000	-0.0R00	-0.0R0J0	-0.09000	-0.08000	-0.0800	-0.08000	-0.0800	-0.0000)))

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A STATIC 334.51025 734.13428 233.81226	33.56421 233.36841 233.07617 232.49222	332,71280 322,65654 332,65675 332,65234 332,65210 332,6560	# SPAN 0.0 10.6430R 10.19685 26.54000 33.08148	63,5237,71,77255,79,482,482,482,482,482,1756,93,492,1756,7990,100,00000	150.70709 186.83635 215.94525 263.1532 306.13569 336.11654 366.68701 394.74414 420.8504 457.14966 468.61499
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¥ >	926	150,38770	153,31310	155,70538	157, 68207	160,62175	162, 39343	163.28696	163.56198		163,21463		163,08271	7	161.30151	7	R ·	0.56521			# 1 0 0 0 0	•	•	1.13646		110/1	•	•	•	4041	7255	1.55654	RADC	2. A245R	1,33144		1.20472	1.32190	1.84650	3.19720	0.41457	-17.52217	-6.42702	-4.40045	10.	585.16309	. 5973	2.73173
>	147.29565	150.38770	153,31210	154,70538	157.68207	160.62175	162,39343	162.2869K	163.56308	3.457	η.	64.1145	63.083		⋖	>>		0.40424	41.T. + 0	0.6744	0.444.0			\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\	Tion of	20101		0.49105		0.40072	(P) (P) (P) (P) (P) (P) (P) (P) (P) (P)	0.49170	FPS	21.06849	Æ	14,00329	11.90725	10.10346	•	F. F7700	4.04645	2.86597	1.90830	1.17484	8	0.48806	σ.	-0.11184
ETA	•	142.84366	145.76409	147.95380	4	152,39287	154,45247	156.14041	157.58072	158,83237	159.00701	160,38611	160.82597	161.22627	61.585	>	•	171440	0.46005		0.47274		750070	0.40010	7007 O	10107 0	•	C 000 0	•	\$1000x	•	7 7	# ADEA	0.0	5.52461	5.3377A	5.21.371	5.11714	10.07594	0.92438	9.84146	9.A0167	0.78893	-	8908	. RO 72	4.80631	4.89266
	140.57603	~	146.56349	148.50229	150.08218	152,58504	154.55809	156.20280	157,61496	158.84308	159.91066	160.38785		263	Œ	5	7 7	. 4	r.	-248 73430	-269.77100	-207 01750	-339.7307E	-369,33862	-306, 42015	-422.12605	16076 777	04464 637	100000	10175 000	08272 007-	r t	-	2 R.B. 00000	288.00000	288.00000	288.00000	288,00000	288.00000	288.00000	284.00000	ď	288,00000	ď	ď	α α	α .	288.00000
A TOTAL	340,23535	340.23535	340,23535	340.23535	340.23535	340,23535	340.23535	340.23535	340.23535	340.23535	340.23535	340.23535	240.22525	340.2353E	340.23535	3	1230 666	248.08072	272,50732								475 00022	485.02285	404 54249	506.07160	517.2215	1600	Tril poess	1.01350	1.01350	0	.01		0		1.01350		_	1.01350	1.01350	1.01350	F	1.01350
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\$	82.2566	184.05661	182.81763	182, 33572	174	178.16656	177.34358	177.5567R	179.29044	179, 79558	177.40785	176. 22648	175,08723	173 66569	171.32347	į	3	0.4540	0.50657	0.43444	7	0.69653	1507	92260	195 88 90	0.04156	0,90122	1,02152	1.03757	0478	. 150.	1.04971	200	-0-10664	-0-21634	-0.24725	-0.20102	-0.3522A	-0.50630	1,48091	7,45100	1.18601	0.649AR	0.46127	0.41700	0.30736	1	72207.0
	246.64062	255.23265	244.50407	230.54132	ADOOR FEE	225.64400	210.05226	14.09741	212.48050	212 50A26	212,61511	212,07256	212,21628	712 58976	717.68500	2		0.48774	•	0.50565	~	16F07-0	0.50132	U. FOOK7	0.50178	0,5023A	71.c05°u	1404.	0.4RC4F	0.48203	4740	•	upo	22,50610	18.47642	F. 5477	13,11104	10.05356	7,28667	4.24790	1.50400	-0.0471c	-7.2P7E()	-F. 7648A	-7.0522A	-8.3CF10	-0. RO741	cz70z°11-
ETA RA	109.42490	118,00581	124.14835	128.86948	132,61275	1205.921	142,37500	145.30127	147.67709	149 44027	_	151 44065	152,08780	152,71532	157.37774	3		0.764.0	0 / 40 40	0.70414	0.6829F	0.66542	0.64009	196690	0.61002	0.60341	0,0000,0	• 50	0.59569	0.50475	0.50344	\$105.	Y AGEA	C.C	5,50512	5,25783	5.12907	£,04440	10.03043	PAGBER	0.79348	C . 70262	C.64757	171757	4.04213	5.01203	5.10477	5,10055
< + t a	110.89253	119.31000	125.17992	129,61102	122.12802	138,53311	142,45117	145.40166	147.48004	140.48147	150.06526	161.66187	152,24238	162,15752	162,92426	3			7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7	1 74.7720	-144.046.04	-165.76100	-100,0010	-740.0776A	-757.1005K	-281.7624E	-202,01272	-217.9787K	774,34375	-330.51270	-336,30460	-241.78638	TOT TEMB	33 P. 19C 95	Opool. Arr	25,100	30004.285	30004 762	174.500R5	334. 500PF	334.65 FC	334. JCORD	325, A000F	338 6900F	340,49995	747, 79080	45.0	347. FCORF
₹ .	מן ערי	۴7.	74K,004F2	164.73107	264. 72102	246.66724	346.46724	346.66724	76727 77637	247,24733	TER ROBGY	740, CR747	271,11604	377, 35577	272.40678	3		r0/r/*/	Trust of	777 - 16536	744.10191	244.43714	747.82084	200.49341	112.61719	072650 212	351.45804	344.12JAS	36c.17456	374.02417	379.40601	סכּונר כשב	TOT DPECC	1.44130	1.46130	1.66130	1.46130	1.44130	1.46130	1.46120	1.46130	•	1.46130	1.44130	1.66130	•	έij	1.64130
ALPHA	40.86371	44.59965	47.14770	48.47025	40.96647	51.00133	43. KRA6K	55.24C05	54° 54376	57.240A6	56.42052	55,693CA	34.07364	53.005AC	52.75BG6	= 3	•	27.72.00 VF.			15045 -461	75 [7 . 0 5]	1 48 46587	1.40.11.021	173, 1740	111. 10422	114.85402	117.18210	110,44336	121.05778	124.61870	127.70857	<□	302.754AR	303,01300	304,98950	306.17163	1005.7051	305,79540	210.54679	211.38222	212.05PAR	313.2480F	316,22705	318,15250	_	355.43656	224.00576
ALPHA RAP	\$000 i • \$ \$	44. 14ROF	48.71097	1 1 X	50, 29475	466	~	55.75044	54° E4424	57.283BO	56. FE434	C Y Y Y B	5	54.306.20	•	*>	335	0 - 1 - 1 - 1 - 1 - 1 - 1 - 1 - 1 - 1 -			00000000	9,000.97	176.77768	176,85640	177,48907	178.26002	78.501	76. F10A	174.09251	173,21115	176,03767	67.caca	90	נמכ	6.1	1,10333	1,21596	•	•	4.07C.	1.20136	•	•	041	¥0ו	2970	Č.	. 3113
A STATIC	- TO / - MOT	40.446.04	350.06641	2149,045	351.504PR	352,52417	012929250	354.11.50	354.12378	354,77210	256.45069	357,53052	15499851	143	2106.19	A CDAR			•	ř	X (5 / 0 / 0 / 0 / 0 / 0 / 0 / 0 / 0 / 0 /	ě,		48.102cF	51.13202	64.79404	75.30270	63.69670	87,79231	c1.8660c	'n	100.00000	=	258.89844	274.71289	289.01562	302,31641	214.90234	234.457PA	360.18709	380.47314	209. 556AP	417.66902	435.160Po	11797.544	452.34470	460,92359	460,49F12
511	0.49.	.14210	09271	•	•	٦,	•	•	٠.	۲.	٠.	۲.	۲.	۲.	ć.	S-V.E. 11F		600	2.000		2 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	0.0000	0.34.17	0.05.44	70.74.0	3.772A1	0.0P?1c	20000	0.09F70	0.17014	0.10457	0.10901	X-VAI UF	770	101	3702	703	2704	705	707	708	07.7	7.	712		9 5 1 2 0	714	۶ زیر

A 24 AC	ROTOR CEC	10-10)/00	(0-310)/40	00/00	Ting/Og1	10/10	TIUT/UT)	d/d	3/3
1.39940	0.87961	1.67471	0.40035	1.63917	1.63017	1,17396	1.17204	1.32817	0.7066222
1.29952	700060	1.91434	0.59512	1.63017	1.63917	1,16771	1.16771	1.40853	0.6978379
1.39963	0.92961	1.89665	0.56166	1.63917	1.63017	1.16380	1.16280	1.46793	0.6020370
1.39954	0.93885	1.04122	0.53622	1.63017	1.63917	1,16215	1.16215	1.51971	0.6892661
1.39954	0.93885	1.99117	0.F1068	1.62017	1.63917	1.16215	1.16215	1. 56173	0.6861295
1.39954	0.94073	2.04013	1.45037	1.62017	1,53017	1.16189	1.16180	1.62645	0.4078005
1.39954	0.94072	2.07314	0.41225	1.63917	1.643017	1,16190	1.16190	1.67037	0.7050573
1.39954	0.94073	2.00751	0.76962	1.63917	1.63017	1.16180	1.161 RO	1.69844	0.7103481
1.39954	009860	7.1177R	0,1110	1.63917	1.62917	1.16250	1,16250	1.71379	7727577
1.29952	0.92055	2.13874	0.20F4F	1.63917	1,63017	1,16562	1,16567	1.71805	0.7422256
1 . 2 994 A	0. R600R	2.16600	0.26505	1.64917	1,63017	1.17604	1.17604	1.71250	0.7413791
1.39945	0.4394F	2.18500	0.25048	1.63917	1.64917	1.18290	1.18200	1.70459	0.7202674
1.39947	0.90990	20100.2	0.23674	1.63017	1.43917	1.19079	1.10028	1.60015	0. 7371948
1.39920	0.779RT	16426.0	0.733R4	1,63917	1.63017	1.19826	1.10076	1.60050	7346014
1.30025	74084	2 2768E	19116	1 42017	1 43017	1 20404	70706	1000	

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VR 43.0600554 37.1500881 31.787808	27.3307800	17.1818737	2,2847052	-7.8069115	-10.8523121	-14.205417R		3>	0.6010131	0.5020042	0.5773647	0.5692067	0.5600024	O. 5557885	• •	0.5625654	0.5659229	0.5677209	0.5726798	57705R	CH a	1.2456202	1.3029736	220008	712041.	1.3844728		407724	1.4103451	1.4060707	1 2771 696	1.3640099	1.3494339	0.4334010
208.53191 206.17610 203.86362	201.48590	196,38573 195,12386 195,24001	106.50766	200-28712	3970	1152	206.25363	3		0.68326	0.73605	4:097.0	0.41202	0.86705	7570	1.02347	1.05413	1.06635	1.0000		9400	-0.14176	-0.10668	-0.26820	-0. 44.046 -0. 46.05	-0.8P03B	-2,72555	1.86558	0.63472	0.26776	0-5-5-0	0.18836	0.16497	0.14500
777.88013 246.67212 258.28140	251.82848	745,145, 745,045, 745,045,045	020-18530	00106-252	19607.450	737.16920	244.4.	XAW	0.58750	0.5849R	0.5770	0.56524	0.55784	0.55425	0.55827	0.56252	0,56540	0.56600	•	0.57740	FPS		10.38315	A.0704F	79166	5.01024	4.007.	7.04140	0.66584	~ (-4.041 B4	-5,12104	-6.37369
113.50478 110.79980 124.62018	128.42230 131.41172	140.17505 140.17505 147.00570	145,00528	147.52972	147.83760	144.06641	144.33377	>	C. PO0 PR	0.74699	0.72162	0.70565	0.64207	0.46479	0.65095	0.65143	0.45665	0.66133	0.67359	0.64108	* ARFA	0.0	5.14710	F.08720	5.04045	10.09122	10.04392	122000	0.01008	0.86153	4,04078	4.97041	4,99793	F.02043
123.98497	124.64156	140,72452 140,72452 143,01360	144.09709	7	101	149.13040	4022	3	•	-119.07680	-159.82330	-176.17671	-204.64737	-259,07935	-281.76660	-301.01704	-314.74820	-220,21704	-330.12540	-334.19180	TOT TEMP	78.0	336.29980	334.19995	336. 69995	774 F9985	114.590RF	234,50¢85	334.79980	338,69995	340,69905	347. TGGRO	345.090RF	347.499A5
367. F2100	366.65601	366.60181 366.60181 366.60181	364,70906	364 B1494	360. PRO65	371.01414	273.67178	3	~	247,59363		265.41738		304.66/174		26 <u>0,</u> 85059		378.28516 282.44010		0	THT PRESS	1.64130	1.66130	1.46130	1.66130	1.46130		1.66130	1.66130	199	661		1.66130	1.66140
47.98125 50.17250 51.77963	52.88315 53.57822	56.46567	50,06001	59.50372	0	,7129	K7.75536		83	169.13134	151.06625	45	136.55112	122,73930	2	115.20116	17.97	122.04304	25.94	9• ء 0	STAT TEMP	200. 75464	300.9709R	302.06470	304.45337	306.17969	307.41138	308-17773	304.70874 300.4080E	311, 98135	313.37476	314. A7866	316.48657	318,10604
	53,1405	56. F1498 56. F1498 57.84673	50.07167	Œ	1439	59.77673 58.26301		×>	203. 84357	207.70987	100.67364	97.4611	195.63266	195,13361	156.59438	104.68355	200.13492	201-10446	203.61546	7 n	STAT PRESS	. 0 AQ	1.12583	1.17671	1,10197	1.21701	1.22426	1.24508	1 26020	1.24394	23	•	.2261	.2192
346, 96720 347, 58672 347, 58672	348,97510	351.39185 351.829595	352,13135	353.91235	354.746KR	355.58374	357.42236	T SPAN	0.0	13,19797	000	25.17931	640	16.75007	66.12RR6	75.03111	83.58615	97.76461	55.97102	000	-	274.35458	287, 20825	310,88965	322,02100	343,19849	363.07202	381.41885	444.55006 614.4.1031	432.61890	0.54	4	6.07	463.69873
747105 0.14966 0.14966 0.18463	0.16658	0.17792 0.19792	0.20681	0.22301	0.2280	0.23206	2400	•	0.0	0.00000	0.01954	3.32471	0.03568	0-05-70	0.366.0	9	٠,	0.09013		0.00412	X-VAL UF	0.0000	3,09466	0.09603	975000		.0926	2,20216	5 1 60 C	000	0.19051	0.09140	02050	0.0000

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-0.23512
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VP 22.7875061 22.8361664	21.8565R9	18,4200835	14.022007	12.0637121	10.4744875	0,0405613	10.2551680	1.254768	12,1177025	14.4047142	008010	17.1201504	M V M	0.6102906	0.4117606	0.4044164	0.4021920	0.5044540	0.5856985	0.5929201	0.5850601	0.58974R7	0.5045871	105204	05744	201140	0.5094023	0.6027605	CHa	1.2647505	1.2043316	116873	1.1343859	1.3475733	1.3472647	1.3701618	********	1.3871021	1. 407CAF	1.3472342	1.2567307	1.3440361	1.3201287	1,3143707
VM 211. 7555 212. 42102	211.50810	207,34000	204.8AF10	204.24594	205,20370	204.98152	208.94010	210.01251	210.60356	211.17.06	213.04017	214.82449	3	0.47578	0.71102	0,74240	0.74857	0.70070	0.84006	0.80360	•	0.00074	1.04504	.0727	1.08322	9500.	5	.1122	2689	-0.41850	-0.41108	- 2	5571	8r915-0-	-1.03587	. 0400	1.19164	6.111.0	0.50273	0.35471	0.29780	0.24954	0.20731	O.1737R
5°, C	267 PER40	252.5244	245.52390	241.14014	238.80850	277.0777	0215.926	240.82147	242.65814	44. 4850	47.4010	750. 976.A2	ΧΛm	. FOA7	440A2	.405	0.5003K	. RG 2	7.785.0	0.58100	0.5843A	۳.	ď	•	N. 59476		5010	0.40085	נטע	A.17770	6.17144	ر د	4.547 A	5,12613	4.17FA7	3.385.80	•	7-75776	crla.	3.0722R	2,29710	010	35	5722
. ACTA AAR 114.34255 120.76085	124,09475	124.4160	135,85573	130.29770	141.97869	143,85005	5	144.29340	146.63480	99000	000	147.18553	> 5	0.80073	0.77653	0.75613	0.73871	FRECT.0	0.70197	0.58821	0.64047	0.67907	0.67874	0.69262	0.68613	660	•	0.70302	And A	0.0	5.21514	4,1,991	£.07605	5.06565	10.12147	10.04799	Q QARG2	0.80257	.0.8050R	0.40245	4.0004	4.05002	4.97115	70406*7
							145,35439	146.32144	146.67841	146.05648	147.15430	147.76860	ī	-100.28917	-124.43701	-144.36801	-164.28404	-181 99613	-711.JOR31	-247.47ROR	-261 550499	-787, 42374	-301,00659	-114.82277	-310,95654	-374.87405	220.21	734.18706	THT TEMP	338. 19085	4.299R	324,19095	Ö	134°4900£	334. 400RE	334.590AF	334° 50085	224.79999	325,49995	338,6000F	340,69995	342.79980	344, 100A5	347. 59095
A TOTAL 368,49194 367,82100	364.02651	366.65601	346,40181	346.401P1	366.601A!	364.70004	367,19702	368. R1494		371.01416	7.741	372.57179	3	08262.286			247.6ACP6			_	_	350 67540	447,2290A	378.441 An	197.00790	387.72R76	300,14624	·C	The DRESS	چ	1.66130	1.46120	1.66130	1.66130	1.66130	1.66130	.661	1.66130	.661	.661	- 6613	.197	1.46120	1.66120
ALPH4 49.49113 51.82010	£9.42002	54.47PAB	56.49172	57.84311	59.2020P	60.40122	61.13570	60.46452	40.21762	40,7800A	49, 11047	α	5	175,96220	166.07019	154.07335	148.97328	144.07628	135,29256	128.19304	122,151.46	117,42052	115,0239	117. A546P	120, 37030	123.07011	126,14145	120.57477	DEAT TEMP	200.76782	0	30C. 479AP	301.82716	303,02393	304.65552	305.71045	306-27124	306.66846	307.44409	300,901 AK	311.46606	313.03222	-	16.370P
AL DHA RAR 40.65597 51.09190	-1	4.6066 5.2110	4.5610	7. 4847	0.2350	70670	1.1649	0.6597	7 A R	9 P 3 D	467F.0	0	>	R 2 C R	1009	7772	4003	5605	24.79	0000	0267	7426	208_4B835	7106	3448	704F	12371	, R	STAT DRESS	1.08036	r.	.127E	.1563	.1723	. 950	2104	21A2	2225	200€	2164	17177	200	2016	4 to 1.
8 STATIC 346.97485 347.23047	47.6452	906	40.8137	50.4182	50.7395	50.9455	51.403R	52.7873	53. F. E4E	E4. F3 C5	54543	1996	NVOV	, ,	765	1.140	7.78	700	5	7.7.7	. C	4.0.4	966	536	73,	7	040	100.00000	=		292,50220		٠.	o	~	·	•		_			œ	55.1	62.1
4n IU 145	^ 4	1621	1767	1832	JOPE	20074	2169	2220	2276	2310	2356	2355		: • c	000	4010	[0]	7.600	6260	0.447	0.52.5	7696	0.70 P	0780	0820	חשלם	0006	0.004.0	- X X	1150	1150	1160	.1157	1150	1150	1150	1115	1150	11150	1157	1150	0.11500	1150	1150

3	CTAD	10-101/00	10-414)/00	מעעמ	100/001	70/10	110/1011
20007		0.000.0	0-00046	1,0000	1.63917	1,00000	1.17296
- 6.5.	0	0.000					
30806	0.0	-0.01926	-0.02499	1.00000	Torge	00000	1.101.1
10062	0.0	-0.03107	-0.03414	1.0000	1.63917	1.00000	1.16389
30068	0.0	-0.03779	-0.03604	1.00000	1.63917	1.00000	1.16215
30004	0.0	-0.04154	-0.03511	1.00000	1.63917	1.00000	1.16215
70 30 E	0.0	-0.04751	-0.03190	1.00000	1.63017	1.00000	1.16180
30006	0.0	-0.05549	-0.03047	1.00000	1.62017	1.00000	1.16180
30006	0.0	-0.06439	-0.0299	1.00000	1.62917	1.00000	1.16180
1000	0.0	-0.36984	-0.07736	1.00000	1.63017	1.00000	1.16250
20806	0.0	-0.07000	-0.02441	1.00000	1.63017	1.00000	1,16562
2006	0-0	-0.06587	-0.C2170	1.00000	1.63017	1.00000	1.17604
36375	0.0	-0.06771	-0.0202	1,00000	1.63917	1.00000	1.18299
3006	0.0	-0.05AR	-0.0180F	1.00,00	1.63017	1.00000	1.19028
30053	0.0	-0.05622	-0.01894	1.3000	1.63017	1.00000	1.19876
1406	0-0	-0.0-496	-0.01766	1.00000	1.63917	1.00000	1.20694

٥	21.6552AR7	4.1007£2	12.2560916	0,8141575	A.6024122	0.1045427	10.0158897	11,0045135	12.0AA775A	14.4522057	16 7037811		DX9.X.1.X.			22. 4027256	* > 1	0.4150856	0.4284024	0.6215301	0.4787740	0.6213937	0.600004	0.5086874	10000000	101/11/0	0. 58. FO33	67141 60	0.5704417	0.5678415	154150	0.5434619	1400F	CH &	1,2745204		1.3019217	1,3154202	7270	3497	1. 2474870	1. 2401274	1.2870468	1. 2884860	1 ARFOGRE		1, 1856878	~	•
7 >	212.65370	217.60302	210.08780	218.31709	216.29039	217.65488	209,52290	207,38715	206,15517	205.09514	201 82007	100.44.00	/ Y / 18 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	۰	50 it .	1140.02711	3	0. 68RJ&	7	707	0.70700	0.81070	0.86530	0.91126	20 CO C		14.00		1.06651	1.07441	1.04130	1.08651	1. AROKO	Juva	4. 506.19	-1.14621	-0.40249	-0. 5056A	-0.54034	-0.62190	-0.78725	-1.12025	-3,17589	6.17544	3,20606	4.12104	-210.09662	÷,	-0.92990
>	274.04224	272 45844	267.0A60A	263.46460	250,17261	251,52251	245.21878	240.33252	724.02017	774. 79644	777 78777	75007 666	-	1416.11	CC041.051	22065.42012	***	0,61190	0.42484	43055	0.42764	0.62000	0.60853	787070	900360		יי מער פר המער פר		0.56848	0.5505j	0.54134	0.54003	0.57607	בטל	F. 944.77	4.267AE	3, 20¢ AB	7.57657	0,070,0	2,45370	7.08630	2,21562	100	4.04075	474754	E. 21445	4,77016	4.44763	7.20072
ACTA AAR	114.62404	121.26490	124,07200	•	120,71227	135.25965	138,02026	141.89235	144.24997	146,11636	147,44512	1.00.40030	07404041	140.1.041	3	150.87020	> 7	40104	. '	0,77749	0.75870	0.74450	0.72042	0.70068	0.48518		0.474.0	V 1000	1,65037	0.65693	•	0.44713	0.64075	V APEA	0.0	F.11778	4.09B74	4.0045A	4.01105	0.82200	0.83575	0,85340	0.84610	04088.9	904000	5.07877	5.14471	5,22919	6, 33482
ATTA	116.74292	121.33545	125.01518	128,001,03	130.73472	135,28494	138.96783	141.93906	44.30394	144,18231	147.75403	140 61475	\	Hollot on to 1	*	151.17181	3	-106.60036	-127,17615	-152,25314	-1 70,05006	-184,11040	-214.59376	-240.42850	-266 61821	306 3676	10160 A 20 CT	0130-06	-414.MINJ2	-324.34361	-120.48504	06849.284-	-330,37666	THT TEMP	135, 60005	235.50000	13K.790AD	775,090RF	135.00000	334.89990	334.79080	334,0000g	335,40005	337,00000	30004 9 F F	330,50085	340,50000	141 . 19990	14001.04r
A TOTAL	147.18021	347.09643	46080.84F	266.87278	366.81924	366.76514	344.71.160	266.8195R	347,1007]	267.0030R	76710 072	7 1 20 1 2 2 2 2		(Int) / (tall)	75.	37U.7R345	3	227,97650	754_47780	767.36365	777. 2 RG70	295,34544	302,10545	218,01722	334 04617	1 10 CHC	070L0 L7E		477. 427AB	7 KO . 1777	2.9270	ď	388.42480	TOT DRESS	1.64130	1.66120	1.66130	1.66120	1.66130	1.46130	.44.	1.66130	1.44130	1.46130	1.44130		1.66130	1.46120	1.66120
AH PHA	50, 01 509	67,02070	F4.70630	55.92230	56,54720	57. 40R50	5P. + 6267	50.40282	60.42673	A0. 8.18.2C	50.81162	FO 00313	ションよう・とつ	00040 *K	Č	75. 62580	fix	177,41502		164.37701	47. 4A70	142, 79072		127,40549	121 45078	04034 744	11/0° 11		- 1			24.0697	128,23613	₹		ä				303.4748F	304° 03 042	304.30908	307. 41900	300.61 ap7	211.47651	312.76123	313.89126	315.05ARR	316,40462
AL OHA OAR	50.14.05	53,00620	24858 73	45,05014	46.56013	11201.77	FR. 40722	FC . 645 74	40.47557	60,96900	50 RO716	00700		C 3 (15) 4 C	67.11970	55. P4697	×>	511.54820	217, 19042	210.74472	214,09639	216.11026	212.45988	200,23836	207.03990	200 776 306	202 1400X		100	•	_	٠.	٠.	STAT PPECS	1.08000	1.10415	1.11987	1.13531	1.15000	1.17603	1.10717	1.21371	1.22580	1.22425	1.241.07	1.24451	1.24976	•	1.24047
A STATIC	244. 72022	346.31274	346.01080	347.48706	348.07300	349.13402	349.97044	35C. 76001	351.42085	252.64111	~	357.61.37	ř	֡֝֝֞֜֝֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֓֡	55.1	4 n 75	760V		4.67947	12.43896	14.756CO	24.45778	25, 34991	45, 46370	55 4BCF2		04. 47404 44 07159		42° (0404	P7.04364	01.36322	5.475	100,00000	-	284.0152P	296.18262	3C7.5A105	714.44287	324 00042	348.00234	367,83268	385,86014	403,12061	419.70100	435,92227	443. 786P7	451,71460		~
SHILE	.1470	0 - 1 572 0	2021.	.1648	.1702	.1805	1903	1007	2006	2172	225	, , , ,	4.77	X	97FC.	.2470	4-18 V-2	C	£ 900	.0122	9710	0235	0325	2620	0.527					.0 RZ 6	9 7 40.	ACOL.	.006,	- V	۲.		.13	-	۲.	-13	13	L.	۴.	-		-13	,سور ا	ري ري	-3

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1.18547	0.000.0	1.67017	1.00,000	0.03.50.0	0.11379	0.0	1.39862
1.18229	0.99329	1.63017	1.00.00	0.03027	0.00010	0.0	1.2 cp 70
1.17017	0.99677	1.63917	1.00000	0.02407	0.07000	0.0	1.30R77
1.17604	1.00000	1.62917	1.00000	0.01901	0.05532	0.0	1.39884
1.17014	1.00387	1.63917	1.0000	0.01127	0.03107	0.0	1.30pok
1.16562	1.00269	1.63017	1.0000	0.00416	0.01031	0.0	1.30001
1,16319	1.00119	1.63017	1.00,000	-0.00440	-0.01037	0.0	1.30003
1.16250	1.00060	1.63017	1.0000	-0.01619	-0.02952	0.0	10006.
1.16285	1.00090	1.62017	1.00.000	-0.02R24	-0.04271	0.0	1,30003
1.16319	1.00090	1.63017	1.00000	-0.03740	-0.04574	0.0	1.39903
1.16454	1.00119	1.63917	1.00000	-0.03806	-0.04165	0.0	1,30002
1.16424	1.00030	1.63917	1.0000	-0.03517	-0.03376	0.0	1.30001
1.16403	0.99762	1.63917	1.00000	-0.02537	-0.02083	0.0	1.30898
1.16562	0.99200	1.62017	1.30300	-0.00091	-0.00042	0.0	rodot • I
(TO/TO)T	T0/T0	1100/00)	04/04	10-0101/00	10-10)/44	FTAD	SAWA?

<u>a</u> >	10,1544930	14,2572396	11,5003842	9.5705700	P. 0474064	7, 403424F	0920780 7	F. R 291 762	A. 743372	12,010,366	14,1899433	14.0957809	15,4984382	00000000000000000000000000000000000000	14.7874208	2		0.6651784	10161919	x/*C'CC'C	0.4142076	0.4011674	0.5742010	0.5744302	0.5700854	O. SEFTORI	0.5564783	0.5300586			0.5061255	1.4047364	כחם	•	3,3405366	1.2454158	1.3484974	1.2715000	1,3784810	1, 2976092	1.3969240	1.4041204	1,4084404	1.4110003	1.4120741	1.4120543	1.4134016	1.4129800	
7	233,40255	227,27419	221,04313	215.44818	210.77641	201.31694	201.62776	200.74247	100,22046	194, 70716	101,37172	187.05648	7405 781	100 12500	176.83041	2	K (7000	9 4 C C C C C	S N S N S N S N S N S N S N S N S N S N	0.44711	0.85260	0.84401	9.00034	0.0504	0.00477	1,02541	1.05633	1.06350	1.0707	1.07635	1.08088	D A D.C	-0.30129	-1.37416	2.1807£	672 TO "U	7.077	1.30640	-4.58543	-1.75744	-1.32710	-1.1704A	-1.01002	-1.00390	-1-10401	-2.00892	~	
>	241.07825	90051.056	240.25426	240.64445	241,41249	741.06277	737 TERK2	724.04624	220.4205F	227,08151	272 B2652	222,10336	220 40851	210 10200	212.00.45	>			•	24.44.0	0.61360	n.60073	10×13×0	0.57475	0.57074	0.54445	0.55544	7 2 7 5 7 D	0.52570	0.51554	0.50421	766076	705	4.70452	1,67061	10640.6	7.544.01	7,1887	1.62065	1.41758	1.4666B	2,62115	3.40AA	4.25730	4.57411	4.87137	4,90319	F.12210	
AFTA AAD	134.3020R	134.26046	134.36705	134.56575	135.16238	127.19718	140.35617	143,00830	145.45479	147.48084	140.31546	150,20901	151,09048	0.000	152, 79001	3		\$1984°C	X	0.44×10	0.64674	0.68855	0.48757	0.67727	0.66545	0.45374	0.64211	0.63050	0.42443	0.61497	19.613.01	1.60031	A A D R A	c.c	4.49041	4.54R74	4.47450	4.78165	0.86521	10.03019	9.95694	0.96957	10.02412	10.10075	4.21399	5.30510	5.40304	5.51867	
Atua	134, 10075	134.41765	134.40452	134.59405	125.9276	137,10845	140.36475	147.10004	145.48774	147.57874	140, 18460	150.28786	151 17633	162 06000	1200.0	ā	7	7777 P 707	3446, 6777	× × × · · · · · · · · · · · · · · · · ·	-212-20699	-211.07656	-717.3052A	-243.34082	-267.37402	-280,40015	-304. 7890£-	4470, 466	-228.31079	-223.0069R	-330,20630	-367.02725	THE TEMB	335,6000F	335.50000	225,20090	225, 0908F	335,00000	UUBBB TEL	224.799AD	225,00000	334,40005	337,00000	338,4900F	330, 5009E	340.50000	341,39000	00062.475	
TV. V	147.18071	247.08547	746,CROOK	246.877R	266.01034	266.76514	344.71049	AKK, PICSR	247,10971	47,0030B	248 A1 404	249, 20517	740,77490	240 2620	70.7934	3	1 6	276.26243	DOCKET OF	(1) \$7 C (1) C	40440 cOr	JUR OJIY	204.22432	316.01460	09455.255	261.34.04	366.16ROS	375.01001		281.44010	294.28127	384.72214	TOT PRECE	1.62240	1.43770	1.64470	1.64400	1.65440	1.66120	1.66130	1.66130	1.66120	1.66130	1.66120	1.64123	1.64130	1.66120	1.66130	
AL PHA	75.54180	•	67.4P32E	63,52391	40° 402 46	56,61903	57,08061	59.058FA	FC. 90796	60.0230	5P. 6P570	57.69733	54.61266	00070*4	54.10840	5	00000	60,0000	00000	,	107.20000	-	"	€.	٠.	_	113,20000	~	_		2	r .	CTAT TEND	306, 93504	306.05313	306.62866	306.33472	306.0507R	306.0341R	306.72070	367, 7002R	300.32544	311.38016	313.82104	315.09765	316.23036	317.55396	314.91248	
b VHdl	F. 4F3E	1.4421	7.5106	2.5464	A704	6. £ 285	7.007	0.0692	טרגע ט	0.0700	7550	7.7409	7050		. 4	>		707		3 1	3366	5221	201,23440	5610	5776	6760,00	4	CC.P449	R7.3573	P3.7462	1040	74.1242	STAT PBESS	1.101	100	203	1,20374	205	7,1	222	234	246	250	111	77.	~	A B B	, O.	
STATI	51.04AA	51.1220	4070	50.7741	50.611R	50.6030	50.9560	51.667	52.4802	53.6474	55,0100	55 7180	56 6167	1000	357.78401				7 T T T T T	0.624.1	7.1720	2. 765P	184	4.3777	5 5 C	3,0127	73,11339	2,1118	:832	1.0502	5.5199	0000	=	8796	5564	BAEC	4060	6745	9052	340P	5740	2002	22.0891	38.6040	46.9107	455.0070R	63,2067	71.4272	
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8FTA BAR 143.89000 145.26767 146.45940 146.80850	1455.70453 1457.851190 1467.54518 1460.24518 150.874605 151.71.335 152.538620	000000	. c u o c c u	0.59493 0.58943 0.59342 4 A O E A 0.0	4.65088 4.65088 4.65510 0.38158 0.55850	10.30618 10.30618 10.30744 5.30350 5.30608 5.51152
AETA 142. A9447 145.28600 146.40246 146.933 146.44881	145.83867 145.888607 145.674601 140.07486 150.01384 151.76127 152.6603	11 8 11 8 2 1 2 1 3 2 4 3 1 3 4 3 1 3 5 1 5 1 5 1 5 1 5 1 5 1 5 1 5 1 5 1 5 1	-200.04164 -280.14649 -200.07369 -311.01870 -332,60210	-343.78401 -343.78401 -240.10116 -707 7540 -335.60005	335,70086 335,00086 334,80090 334,70980	335.60005 337.00000 337.00000 340.50000 341.30000
A TOTAL 367.19021 167.08662 366.08006 166.07178	266.71060 266.71060 266.71060 267.10071 367.00308 368.81404 360.7500 370.77400	2 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	361.65200 365.02000 367.05660 373.65660 373.65660	180.00674 384.17554 387.11270 107.02550 1.61980	1.61980 1.62120 1.62670 1.64610	1.66130 1.66130 1.66130 1.66130 1.66130 1.66130
AL DHA 80, 00007 80, 00007 87, 76633 83, 47151	74, 75,067 62,57267 58,20807 58,76807 57,6610 64,6617 56,66778		79.50000 c9.6co9c lif.7co9c 112.50000 115.20000	170, 2000 123, 00000 176, 00000 274, TEMP 313, 50260 313, 88940	314.10303 314.10156 313.84307 313.10114 312.31641	
A! DHA BAB PO. COCCT RO. COCCT RC. COCCT RT. TTOOS	74.7477 68.05736 58.05736 58.7376 58.7370 57.63718 56.70050 68.67361		197.0780 191.67374 185.21.84 186.26644 181.4002	176.87000 176.78155 165.90082 5787 PRESS 1.27432	1.28831 1.2021 1.20411 1.20411 1.30005	
A STATIC 254.04204 355.06689 355.19189 355.19312	354, 64866 354,18350 354,18350 354,0014 356,00283 356,71582 357,38208	7 × × × × × × × × × × × × × × × × × × ×	אוריים יון החוץ יוני	90. F5355 90. F5355 90. 650000 100.00000 100. 00000	311.62451 322.06177 322.18262 351.65430 370.44165	405, 14037 474, 41373 441, 45972 449, 90210 458, 33301 466, 78491
7US 503 657 719	0.18200 0.01120 0.20120 0.21056 0.22666 0.23285	3.74500 5.4 AI UE 0.00 72 0.0112 0.01160 0.02193	0.04174 0.04174 0.05120 0.05056 0.07666	0.08722 0.08722 0.09530 0.09530 0.17500 0.17500	750 750 750 750 750	0.17500 0.17500 0.17500 0.17500 0.17500

STAGE	Ecc no/	16-101/	(0-010)/0J	00/00	1 50077	10,710	1 14662	9/9	V/V 0.046423	
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) C	Ċ	19462	0.10798	0.08486	1.59822	1.0000	-	1.07164	0.4598281	
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0	Ö	19812	0.12154	0.98326	1,50503		_	1.07346	0.8549863	
0	o	.18342	0.11458	0.08501		1.00000	1.16285	.06.Al	0.8672123	
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>			104,82182							182,100	175.72		160 35 001	157.24	144, 28591	3	1		-				1,17154			1,12541	1.107	1.0853	1.05007	1.0550	1.0405	7089	-28,1024K	-1.54]							4.350	2.022	2,833	-16.14482	-0.0-	12002-0-	-0.00
>	107,35878	104.87273	105,83182	104.51860	102.13246	190,60461	188.63507	189,51052	191,60361	194.18201	104.43512	107 1/013	106 69601	105,23505	101, 78897	>>>			17075	0.54550	0.54142	0.53358	0.52741	0.57616	0.51PER	0.50822	0.48050	0.47045	0.44200	0.42772		100	-0.280A1	0.30746	0.82014	1.32049	1.845277	7.040°C	3.75163	4.12675	£ 200 7	7701 b. F	2.70770	4.55A0A	3.24856	001.	7 86101
AETA AAR	146.74562	146.84226	147,02412	148,06201	140,04741	151.49385	153.18230	152.07094	152,83064	152,10037	153,700%	154 24872	55.2341	156.34164	157.68350	3	u	u		0.54573	ď		٠.	400F70	0.53657	0.54211	0.54835	0.54967	0.54760	0.54255	•	A AOFA		£15879	4.4R04R	4.10176	4.72954	9.52034	0.51226	9.64176	702UL	0.47700	10.14391	F. 24568	5.52012	F. A2510	30770
9ETA	145.74500	146.94358	147.92680	148.06964	140,04541	151.72537	152.23184	153,00026	152,08754	153,15173	153,74760	154 20102	155,27166	154,37263	157.70834	5	9	11016 1061	-312,47534	-377 JEOF?	-332 . 7404R	-357,90030	-373,15112	-270,01172	-361.45796	-358,96362	-355,54FOO	-250.60A15	-245,42578	47.7	-353,95532	THE TEMP	335. 6900F	335.50000	DRODC SEE	335,00085	335,00000	334. 99090	334.790AD	335,0000	315,49005	247,00000	734,69C0F	330° 40084	340.50000	341.39090	00000
A TOTAL	16001.795	247.09643	364.58004	346.P737R	366.81034	366.76514	346.71069	3KK, Aloka	367, 10971	47.0070F	368 R1404	340 20517	360,77400	370,25293	770.78345	3	350 43044	350 050 26	24P. 740K2	377, 27107	195,50200	401, 56460	419.12109	415.16333	406,51221	402,51543	304.59AAA	389.24660	280.41.00	10c44°01c	06214.585	Sudd FUE	1.61980	1.41080	1. FIGAN	1.61980	1.41080	1.41089	1.41080	1.62570	1.63500	1.64340	1.45140	1.45590	1.45950	1.44120	
AL PHA	4000000	49, ccooz	75000°08	89,00007	15000° 58.	80° cac 61	40,00007	63.45093	75.41247	69.64957	63.40277	50,030BE	54-06687	2474	0116.	17				0.0	°0	0.0	0.0			•	P7.7000	5	-	•	r	STAT TEMP	316,35571	316,24002	316,250cR	316.20403	214.47102	316.95357	317.12427	217.16040	317.44460	314,27246	214,53931	320, 30371	321.20321	122.47cog	337 11170
ALPHA RAD	0000	0000	6 206	906	6060	0000	6060.0	82 .49572	5461	500	63.45070	0.0704	4, 1122	F1.78864	•	*	. ה	24 9 42	٠.	104.46703	63,0	190,35769	AR. 2317	187.79657	JRE. 16864	181,70500	175.35432	68.79	59	5,5	145.13713	•	1.31545	•	•	1.22303	•	1.33380	•	1. 34271	•	1.34485	カッサ	.3485	4576	3.60	2707
STATIC	56.4531	56,3091	356.40430	56.4367	56.5312	468	700	356.01895	874	357.53467	358.23056	503	50,1945	50.8471	~	NAGA		6.00172	· u	17,66103	23,19554	34.8421R	44.00900	53,70523	63,02431	72,11116	81.08026	85.60327	90.26210	•	1 00. 00000	=	289.81177	301.34937	312.47534	0 324,25GF?	237.7404R	353.90039	373.15112	22115.12	404.15706	426.36377	443.345cF	451.90820	460.72583	440. A35c4	470 15557
2 An 1115	0.15000	3.1 4407	0.15173	0.16721	0.17273	0.18217	0.10313	3.23263	3.21176	0.22067	953660	0.23380	7.2266	3,24217	0.24800	S-VAI 115	; i	9	7	173	227	158	431	3.05262	517	706	302	0.08380	400	160	0 8	-VAL !!	5.0	5.0	0.20	S C	0 K O	0.0	0 20	S C	C L	0 20	50	50	0.0	0.10500	200

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٠ د ه	0.0	0.11906	č	1.00000	1.50022	1.00000	1.16562	1.0322B	0.0335256	
Œ	0.0	0.10720	Ö	. 30,000	1.59822	1,00000	1,76403	1 02441	063636 V	
10	0.0	AC 500 J	ò	00000	1,50822	1,0000	1.16424	1.02619	06 30 20 0	i
60	0.0	0.00395	c	0.99914	1,50822	1.00000	1.16354	1.02303	0.0440780	
£000£*1	0.0	0.09 P49	0,0255!	0,00574	1.50A22	1.00000	1,16210	1.02531	0. 0258080	
50	0.0	0.11710	Ö	0.08CP6	1.50222	1.00000	1,16295	1.03104	0.0117562	
40	0.0	0.13920	Ö	0.04402	1.59822	1.00000	1,16250	7.02817	0.8866530	
۴.0	0.0	0.15311	ċ	ACE8360	1.40502	1.00000	1.16210	1.04391	0.8769010	
5	0.0	0.15455	c	0.99417	1.61322	1,00000	1,16562	1.04504	0.8785602	ì
90	0.0	0.14577	ċ	CC08000	1,67151	00000	1,17014	1.04183	0.8046836	
4	0.0	0.13403	c	0.00414	1.62960	1.00000	1.17604	1,02721	0.0127277	
11	0.0	0.12800	Ċ	0.09460	1.62375	1.00000	1,17017	1.03523	0.0711210	•
70	0.0	0.12725	o	0,0991	1,63641	1.0000	1,18229	1.72628	0.0251244	
62	0.0	0.12383	Ö	1.00000	1.63017	1.0000	1.18547	1.02543	0.0251766	
72	0.0	0.14968	Ċ	טטטטט 1	1 63017		0000	7000		

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α>	0.2695813	1.6701412	2.0161844	3.9193983	4.6406964	c. c156155	6.070010	7,1678419	P. 7239094	0.7296412	9, 51 20106	8,4445004	4 0124162	2,2256823	-2.408330a		¥ > ¥	0.5240510	0.5247575	0.5252102	0.5252679	0.5247424	0.5210141	0.5141280	0.5055910	0.4970944	0.4807121	0.4853256	0.4417162	6720343	0.4660638	0.4573948	CI a	1.4703398	1.4707003	1.4712447	1.4720840	1.4720071	1.4760723	1.4915187	1.4467802	1.489710R	1.4894K7K	1.4PP4R7E	1 4879255	1.4871950	1.4843768	1.484970K
3	187,34756	187.54890	187.64789	187,61234	187,40672	186.11627	183,75307	180.90147	178.10008	175,83438	174,90514	173,92275	171 94502	168,00041	145.59362		3	0.96525	0.90407	1.02102	1.04846	1.07470	1.12-14	1.16819	1.21060	1,25141	1.29054	1, 29126	1.27927	1.26379	1.24626	1.23752	JUVa	4.03920	4.00601	A. 28051	-27,92952	-2,87601	-1.11580	-0.69820	-0.507A	-0.61022	-0.42407	-0.61633	-0.64604	-0.71100	-0. AUT74	76020-0-
>	187.34254	197.54090	187.64789	187.61234	187.40472	186,11627	182.75207	140.00147	47001. 47.	175.83638	175.54714	175,05252	176,40203	177,02072	177.12463		×>>	0.57405	0.52474	0.52516	0.52515	0.52458	0.57079	0.51285	0.50518	0,49640	0.4PB46	0.48461	0.48115	0.47564	0.44607	0.45735	V du	0.08244	0.51023	0.80045	1,10705	1.42170	1.60873	1.805,84	2.27082	2.80623	2,17530	3,11750	2. TR467	2.00420	0.75400	-0.43781
RETA RAD	147.11740	148.13605	140,07339	140.04717	150.77319	152,36145	153,89922	155,31696	156,59557	157,72353	157.92201	157.96122	157.87714	157,99472	150,30075	;	≩	0.52405	0.52474	0.57521	165650	0.52474	0.52101	0.51413	0.50558	0.49708	0.48071	•	0.48762	4.	1,49941	50887°U	A AUFA	0.0	4.04717	4.94003	080£8.4	4.R4103	9.70365	0.77153	0.47750	0.99477	10.13369	10,27498	5.14215	5.19140	5.25300	5,35722
RETA	147.11745	149.12700	140.07646	149.95259	150.78073	152.37178	152,00163	155,33406	154.42045	157,75441	157,05242	157,98483	157,88931	157,00646	154, 11082	•	3	-280.R1177	-201.74390	-313, 20703	-224.26219	-334.05728	-355°42505	-374.90945	-343.61544	-411.68678	-420.23071	-431.22413	-427.24463	-422, 96729	ر م	-416.55542	Trit TEMB	325,69995	235,50000	335,20080	375,00045	325,00000	334.80000	334. 790AD	335,00000	135, 4000F	337.00000	338,69095	330.50085	240,50000	141.30000	06006.546
A TOTAL	347.19921	367.09642	366.08096	346.8737R	344.81034	366,74514	246.71069	166. A1059	147,10971	367.0030A	369. A1404	349.20517	149.77490	770.25202	370,79345	;	*	145.10205	FU084 33E	265.11694	174.62646	283,010,582	401.205A1	417,51831	433,10556	448.50473	463.84085	464.35425	441.25073	456.59154	450.70712	448.2C9RO	TOT DRESS	1.61090	1.41990	1.61080	1.61980	1.41080	1.61080	1.6.10.80	1.61080	1.61980	1.61080	1.62530	1.62940	1.63360	1.43770	1.64060
AL PHA	40.0004	89, car 97	40,0000	Ö	000	ō	000	000	89.09CQ7	80.00007	45.09097	81,06378		72 577RE	69,30112		î,	0.0	0.0	0.0	0.0	٥.	0.0	0.0	٥•٥	0.0	0•0	15,00000	27.29090	30, 79000	23.00000	06605 * 69	STAT TEND	318.26563	318,02030	117.80981	317.61597	317.55420	317.69336	31 R. 02759	318.7447R	710.92847	221.64429	323.39746	324,22827	325.03638	25.94	6
ALPHA BAR	80°00001	80.00cq7	00000	40.0007	80.00007	80° 0000 1	80° 0000 1	89,09097	Ac. docc 7	80°00001	85.00810		76.96732	72, 57022	69.30307	3	Y > 1	187,36737	E (Œ:	187.34002	186.03453	183.6F24R	180.75940	77.0763	175.56442	_	173.61761	•	168.80514	65.6761	+ 50	.3435	8 2 7 2 8	.3424	£25E •	8 4 7 8	1.34632	1622.	•360₽	.369	7566	PAPOL	.3850	.3A7	1,39050	•
A STATIC	357.52734	357.40098	357, 29140	745	357.1403R	357.21924	257.40723	357.8090A	259.47046	423	360.38721		_	761, 71924	-	į		0	6.30088	x2406.71	14261.81	23.83966	34.64812	44.93684	F4. 81644	4	3.6274	2.6087	9.09	1.3420	95.66	100.00000		289. A1177	301-74390	213.20703	324.26318	334.0572R	355.42505	374.00845	393,6154R	411.68628	14062.045	446.23413	454.54468	462.76733	ξ.	479.15552
PANIUS	-	٦.	7	٦.	٦.	덖	۲,	۲.	۲.		٠.	٠.	٠.		٠.	:	א א בי	0.0	0.0017	0.01211	0.01783	3.02436	90260.0	0.04404	0.0 8372	0.06307	0.07215	0.08006	3.08526	0.08357	0.00375	0.09ª00	3	2	7	2	2	~		_	7	~	_		2	2	0.21 .00	_

ANAAA	STAGE FFF	10/101							
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4080K	0.0	0.09220	0.02400	1.00000	1.59822	1,0000	1 16563		
3000		,,,,,,,					, GLUT • 1	1.00/153	#Z47.54
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10000						1.0000	ATCOTOT	1.001204	0.9703533
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7000E	0.0	0.04785		00000				2001	T2Chart *0
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£0505	0.0	0.06290		0.00576	1.59822	1,0000	1 16210	000	
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3CB 9.4	•					C.O.O.O. • T	*	1.02274	0.9055132
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\$ T.L.) N. +	•	*0*.0*0		0.cg754	1.61975	1,0000	Coool	73710 1	

a >	-0.0647919	0.4271557	0.0159402	1.4021158	1. RO47868	2. R065958	2.4068441	7.4361267	2. RR37R43	1,9510231	1.0704622	0.7745374	0.4857477	0.4971714	647364	*/*	7107617	01272750	0.5122166	0.5111508	0. 5008800	0 E036430	0.444.01.0	7000CCS 0	0 500225	0.4966415	0.4028895	401154	0.4898329	0.4803142	0.48096AB	C 3	1.477789	1,4790812	480617	1,4973351	1.4836003	1.4858904	1.4878807	1.4885507	1.4874930	1.4842740	1.4704389	1,4767209	1.473777	4	1.4654570
>	183, 97992	182.62338	184.27248	182, 92623	182.36720	181.50876	180.74690	180.05258	179,26534	179. 34026	177,53531	177,17015	176.94550	30.50	177.47614	3	20030	0.08762	1.01575	1.04207	1.06800	1 11024	1 16672	1 21 28 4	1 25426	1 20698	1.33560	1.35422	1,37304		4104	9 800	-10.34649	-4.67942	-7.11497	-2.52521	-2.37 PK4	-2,47211	01019-6-	-7.14583	-1.4503A	-1,17021	-1.14488	800	-1.95147	717	1.58325
>	•	r	103.27748	187.87672	182.26720	191.50826	180.74690	180.05258	170,26534	178.3A026		177,17015	174 04550	174,00503	177.4714	×	0.51370	• (•	0.50986	0.50738	0.50520	00503.0	0.50017	0.49661	0.40788	0.40115	0.48083	1 C C R C 31	0.4809A	u C	-n.02019	0.142A	•	17067.0	•	D. APSOR	1.09002	1.00250	0.02174	N. 6765R	U. 24547	0.25113	90,000	0.22569	0,20415
AFTA BAD	147.60567	148.69096	149.69402	150.62505	151.48521	153.01578	154,37685	154,40312	156.52426	157.48332	158,34163	158.73442	150,00014	150,41704	150.6756K	>*	0.51270	0.51322	156120	0.51116	0.50040	0.50744	0.50528	0.50309	0.50023	0.49664	0.40290	0.49115	0.48983	490	.4800	A AUFA	٠,	196	70,5	£ 900 4	70620.7	O.88740	0.01735	0.94616	0.08461	19.040#2	10,11092	F.083K]	5,00042	5.11.27	f.11A65
Arna	147,60567	148.60101	140,46435	150, 62574	151.48653	153.01857		155.40706		147.48453	154, 341 06	Ç	ď	150,41721	7.	3	-280.81177	401.01650	-212 55850	-274. TOKK3	-125.67261	-256-67339	-274.18350	-304.04387	-412,05801	-430.2044R	-447,07202	-455,27344	-462,35449	-471.21641	-470,15552	TOT TRAD	336,49095	335,50000	414,20094	135. JOORE	335,00000	00006 * 752	08001.4FF	335,00000	335,60005	337,00000	72009. REC	330 GCORE	340.50000	4	
A TOTAL	16091.745	267.08642	366.09096	746.A737A	366.81024	344.75F14	346.71060	166.9105A	367,10071	367.00108	368. R1404	360,29517	ď	70.7	370.78345	3	747.274R	253, 17622	243,10002	277.1170	382.01.204	400,00705	417, 25303	474,06976	450.18921	445,80371	441.03247	488.53149	406,90707	203.45450	510.04753	TOT DRECK	1.61080	1.41080	1.61080	1.61380	1.61980	1.61980	.619	610	619	3	٠٤٠	1.41980	٠٤	.19•	1.61980
AHO IA	80, 90007	Ad. occo7	80° 000 61	40.000	. 80° 000C7	80° c 000 1	go 00001	40°00°1	89.0007	89,00007	AO OCOG7	ga aaca7	80,00097	49,00001	40° 00'04	î>	, c	0.0	C C	0.0	0,0			0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	STAT TEMP		319.75122	214.41597	318.4965B	318.47cop	114,53491	318,57178	31 A. 40649	310, 73755	321.19620	223°04883	224.01514	•	8 6 5 B	226.76782
ALDHA RAP		2000 0	200.0	0.000.0	·	0000	0,000.0	č	000000	Ac.acco	30000	40.0000	Ö	80.crc97	40°cacd1	× >	183,87081	183,63787	182,27010	187.870RF	Œ	α	Č			178,36760	177,62270	177.16844	176.94426	176,00455	177.47501	STAT PRESS	1,25304		354	757E.	. 25F	٠. م	.760ª	1.36285	1.36546	1. 36973	7	1.97277	3740	2752	1.27482
A STATIC	Œ	357. A0615	2-1343	357.66643	7.6604	357,60180	_		358,36228	359.17310	74201.0AF	360,7219	361.23657	161.72241	362,72070	Y SPAN	0.0	6.30100	12,52078	18.47427	24.21767	35.2026R	45.61170	55,53004	65.03346	74.18006	83.05183			۳.	1 00.00000	=	289.83177	01.0	713.55pra	324.79663	335.47261	256.47330	376.183EQ	304.06287	412.05801	430.29468	47	r v	63.	7	479.15552
PAPIUS	0.1 5000	3.15624	2.16220	0.16810	0.17272	0.18450	0.10470	3.20442	0.21273	3.22271	08 ir c ° C	0.23554	0.23682	20.242.0	0.2420	C-VAI 115	c	.0062	0.0122	0.01410	0.02373	0596000	0.04470	3.05442	F L E 9 U U	17570.0	0.08130	0.0AFF4	0.08982	0.0000	0°0000	x-VAL (1F	0007400	0.24000	0.24700	00046.0	0.24000	0.24000	0.24070	0.24330	0.027.0	0.24000	0.24000	0.24200		0.24000	2

						1	:																• •			!!!	i					
	2/2	0.0814117	0.9766828	0.0744894	0.9731091	0.9752412	0.9836401	0.0053074	1.0060339	1.0144787	1.0113249	1.0069141	1.0025692	0.0000000	1.001001	•							i -		•	•						
6		1.00709	1.00892	1.00076	1.01026	1.00042	1.00601	1.00167	To. 60 .0	0.00513	99.28	0.99182	0.0000	0.08913	0.98667																	
		1.16562	1.16424	1.16354	1.16319	1.16285	-,	1.16319	-	•	1.17604	7	∹	~	1.18889																	
6 + 7 6 +		000000	1.00000	1.00000	1.00000	•	1.00000	1.00000	1 • 00000		1.00000			•	1.00000			٠														
1100000		1.50472	• •	1.50822	1.59822	1. 50822	1. 50977	22kps-1	•	•	•	•	•	1.59922	•																	
	7		1.0000	•	•	•	•	•	•	•	•	•	•	0800	0.98732																	
10-010/00	Ξ,	0.0000	0.00047	0.00071	090000	£1.00°0	0.00453	0.0011	16100.0-	-0.002A1	-0.00411	-0.00485	-0.0056R	-0.00689	-0.00862																	
10-101/00	41.41.4	0.03640	0.04318	0.04720	0.04973	1845000	0.03045	0.00.0	KS 110 * 0 -	-0.07744	-0.04060	-0.04439	-0.05250	-0.06113	-0.07514																	
2440	L !'	•	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0													•				
437	4 6 6	1 30808	10002.1	1.10003	1.30903	1.30c0	1.39004	1.30003	10256-1	10401	1.30884	1.30877	1.30070	1.39962	1.30854	٠									:		:			,		

° >	0.0212545	0.4092136	0. 4832701	1.1569118	1. 3302326	1.4764090	1.5327530	1.4952523	1.8093036	1.0769096	1.679931	1, 3350737	0.9471472	0.2867502	-0.21481RR	2		2402.00.00		0.5056766	0.5052174	0.5047350	0.5022806	0.5001028	0.4988260	0.4086258	0.4094053	0.4000735	0.5003830	0.5004000	0.4009000	CH 0	1,4830468	1.4847740	1,4855337	1.4064120	1.4840480	4,489,968	1.4900188	1.4004107	1.4884706	1 4828787	1.4748869	1.4705811	1.4446048	1,4475702	459604
2	180.04040	180,05815	180,04122	180.80214	180.81126	180.41502	179,72661	179,06514	178.79394	170.05770	170,77165	180,20313	180,58073	180,82242	180.01104	3	0 0	ָרָ <u>י</u>	*****	1,02063	1.06647	1,11776	1.16665	1.2127	10256	1,70074	1.33970		•	1.39778	1.41574	7000	20.05042	21.19242	33.41321	-73.01605	-11.4PR54	-3,5A062	-2,24520	-7.54642	-4.44.046	12,20474	4.11047	4.23980	0 4 4 4 0	-R. 23304	-2.62507
>	180.04040	180.05815	190.04122	180.80214	180.91126	180.41502	170,72661	170.06514	178.78394	170.04770	: 70, 77165	180.2021	180.58023	CACCP. DA.	190,01106	>>	10303	2023	77204	0.50547	0.50530	0.50422	0.50227	710050	040400	0.49860	0.4002A	90007°U	0.5003A	0.50040	0.0000	701	F7 3000 0	0.15775	0,27060	0.26646	0.42153	0.469BB	0.48864	0.53077	0.40440	0.42250	0.53544	0.42479	O. JERRO	0.000a4	-0.06803
M,	48.0100		150.03072	150.0020	151.71733	157.19405	154,40325	155,64191	156,61051	157.42185	154,11330	158.42000	158.71735	150.01488	150, 21523	N.	ď	•	•	0.50548	•	0.50423			0.49893		0.49941	100000	0.50038	0.50040	0.40000	A AP FA	0.0	4.07090	4.94827	4.94603	4.04640	0.03865	0.05685	0.08308	10.01100	10.03476	10.05219	5.03082	5.03376	σ.	5.0475
AFAR	144.010004	140.07350	150.03105	150.0086	151.71790	153.18571	154,40417	155.64294	154.42070	157.43306	154,11417	158.42053	159.71751	150.01488	140,21423	5	-280.01177	-202 04102	212 7882	-325,10864	-336.0468B	-356.03042	-376,69763	-304.515A7	-413,53174	425 9.052-	-447.49609	-455.60645	01077,524	4	-470.1EE52	TOT TEMP	336,69995	335.50000	135,29080	325,000RF	335,00000	34. 99990	124.79980	00000°562	332,49095	23.7.00000	334,60095	330,59985	340,50000	341.30000	0000L.C4F
A TOTAL	267.19702	347.0PP97	266.09071	266.97231	366.A1A36	366.76416	366.7099F	366.91936	367.19702	367. AGG17	348.81404	369.2CBR3	360.78723	170,76480	a2004°31k	3	341 - 66870	٠ ٥	362 21024	772,04517	70,000	300,03657				446,54152	442.25562	480.04022	407.50854	04	512.17000	TOT DOESS	610	1.61990	1.61080	1.61080	1.61980	1.61040	1.61080	1.41980	1.61080	1.41980	1.41080	1.61980	1.61980	1.61940	1.61080
ALPHA	80.00c07	go daca7	80.0000	90° 0000 1	1650c.ph	40°occo1	ge roco1	80°00001	89.00007	80° 00001	80° 00007	80.000C7	89.90007	40.0007	Po.coca7	5				0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	STAT TEMD	119,43506	319,23511	319,03760	31 A. P4610	31.8.76074	318.73145	31 4. 75464	310.07300	310. A2397	06920.165	322.65210	323.47607	324,31006	325.16772	726.1530R
L PHA B	ů	0.000.0	202.2	40.00004	90° cor 91	7 poop . 28	8c.corc7	Ro. acco.	Ro. dord 7	80.90017	Ac.oocd7	89.00cg7	80° coc d 1	40.00.04	80,00007	× >	1 80, 06040	180.05746		18C. 88843		180.40089	175.72006	170,04721	178.77277	17c.04588	170,76270	180,10917	80.5782	õ		STAT PORCE	.3609	3607	505	1.34067	604	1.361RI	1. 16258	1.36550	1.34674	1,16692		7	Ľ	۳	1.36592
A STATIC	258.19116	358.080?2	357,07070	357.86450	357.81714	357.40127	257.91470	367.99210	358.4091B	359,10254	350.07144	360,42520	360. PA402	261,35547	361.89624	NAGA	•	6.45834	12,44200	18.64024	24.41679	35.44542	45.48213	55. 82346	65.33473	74.47752	83.2781E	•	÷	95.916P	100,00000	=	289. A1177	302.04102	313.78833	325.10R64	335.0468F	376.43042	776. 69263	395.51547	413,52174	430.83398	447.45600	Š	463.57010	471.474PO	479.15552
11.15	000	7 4	3,16741	5 R 2	200	747	3.10434	747	40	200	116	35.0	0	0.24400	0446.	4-VAI 115	0.0							0.05471				0.08583	0.08004	0.00400	00.000	X-VAL UF	0.26000	0.24000	0.26000	0.26090	0.24000	0.25000	0.26000	0.26000	0.26330	00092*0	00092.0	0.26000	0.26000	0.25000	0,24000
																							;	! I		:			•			!								:						; ; Δ _	-5

	dΨ	10-401/00	10-010)/00	ud/lid	1 (04/ 04)	TU/TT	TIOT/OT)
	C	0.02056	92400	1,00000	1.50922	1.00000	1.16562
	C	0.02714	0.09617	1.00000	1.59922	1.00000	1.16493
	c	0.0177	0.00502	1.00000	1.59822	1.00000	1.16424
		0.01975	0.00389	1.00000	1.50822	1.00000	1.16354
		0.01504	0.00704	00000	1.59822	1.00000	1.16310
30003		0.01126	0,00142	00000	1.50922	1.00000	1.16785
	· c	0.01057	0.00157	1.00000	1.50822	1.00000	1.16250
	· C	8,010,0	0.00132	1.00000	1.50072	1.00000	1.16319
		0.00505	0.00058	1.00000	1.50822	1.00000	1.16562
	2 0	-0.00715	-0.00075	1.00000	1.59922	1.00000	1.17014
	· C	F8FC0-0-	-0.00226	1.0000	1.50822	1.00000	1,17404
	2 0	-0.02745	-0.00293	1,00000	1.50922	1.00000	1.17017
	2 0	-0.03001	75500-0-	1.00000	1.59822	1.00000	1.18229
0.0	. C	-0.04108	-0.00340	1.00.000	1.50222	1.00000	1.18542
	· c	-0.03675	-0.00204	1,0000	1.50822	1.00000	1.18880

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SOLUVO	A STATIC	ALDHA RAD	AL PHA	ATITAL		45.0 5.00 G		F 0 F	2 6	1
0.15000	358.31177	80,000	70000 0K	367.19702	144.10073		170-74006	174, 7900	0.0	:
0.15424	354,20620	C	80° 00001	167.0APR7	140.75345	40.244	02.02.01	62207 ° 07 1	1018701	
.1624	354.10083	σ	40.000.68	146.08071	150.21666	150.21661	170,64723	170.64723	0.3244453	
1683	357,09512	8c. 99co7	49, 99007	146.4721	151.00401	151.09496	170.50204	170,59204	0.4734954	
1740	357.94239	Ç	80°0004	364.91836	151.89650	51.	170.56453	170.56453	0.60A2769	
7	357.88965	0000	80.00001	146.74414	153, 31483	53.3	170,43431	179.53531	0. F13277A	
1061	357, A36c1	Ac. 00007	7000 cod	346.70c96	154.53777	5.4°	170.50606		0. 4830458	
0.20484	247,04262	40,000,00	Ro. caca7	366. A1 B26	155.50660	155.59630	170.56122	170.56122	0.4030497	
2141	358,31201	70000 og	40.0004	20701.775	154.51628	156,51617	170,75577	170,75577	0.6153533	
2230	358.00707	80 cocc3	89° c 9c 97	367. R9917	157.31825	157.31821	180.11363	180,11363	0.3936383	
7186	359,89063	0 000	80° c 90 97	368.91494	159,02696	158,02690	180.58224	180.58224	0.2070187	
2356	340,34270	Ü	40 0000	369.20883	154,35393	154.35348	140.82010	190,82910	0.1404766	
0000	740 83647	40,0000	89,09007	269.78723	158.44544	158,66544	191.07555	101.17555	0.0017400	
2 44.0	341 30542	90, COCO 7	40,000,00	270,24483	154.06743	158.96263	191, 12184	181.32184	0.0407150	
7	261. P.7 P.P.	80,0CCG7	٠,	370. ROO 20	40,2437	150,24377		٠.	0.0	ļ
:	•	•								
4-VAI 11F	NAGS	×>	۸n	3	3	3	×^*	₹	# V #	
0-0	0.0	170,75004	0.0	341.03394	-280. A1177	0.50148		0.95178	0.5014834	
•	4.48668	170,7071	0.0	251,50105	-202-002-00	0.50147	0.50.47	0.08178	0.5016726	
0124	12,71817	170.44694	0.0	361.66553	-317. AC307	0.50167	n.50167	1.00005	0.5036666	
0182	18.72217	170,50142	0.0	371.54434	-325,26147	0.50164	0.50166	1.03796	0.5014606	;
•	24.52214	170, 56349	0.0	241.1867R	1246.365-	0.50166	0.50165	1.06404	0.5016576	
0	35.58640	10	0.0	300 17464	-487,19312	0.50165	0.50164	1.11703	0.5016499	
	46.03524	179,50388	0.0	417,57304	-276.0777B	0.50164	0.50144	1,16642	0.5016420	1
3.354R4	55.96286	170.55042	0.0	434,60352	-295,77515	0.50165	0.50164	1.21417	0.5014480	
•	65,44553	179.75471	0.0	441.00252	-413,7200R	0.50167	0.40147	1.25904	0.5014739	
0.07305	74.54575	180,11319	0.0	447.09447	-430,96069	0.50171	0.50171	1.30108	0.5017133	•
Ö	93,313ck	186.58712	0.0	482,67012	-447.56250	0.50177	•		0.4017498	
•	87.58604	3	0.0	490.22168	-455,65127	0.50140	•	1.36036	0.5017073	
.0	91. 78867	1.07FF	0.0	497,71606	-463.60864	.501ª	•	25075.	0.5019244	:
906.	95,92534	α,	0.0		•	r.	. F01 a	5	0.5018519	
0 9 P	1 00. 00000	_	0.0	£12,41260	-479,15552	0.50188	0.50189	1.4]619	0.4018820	
3114717	Ξ	CTAT DBECC	GMST TATA	POTE TOT	TOT TRMP	A APFA	n C	PADC	CIa	
0.0000	280,81177	1.36414	10.6501	61080	•			1000000,56250	1.4864540	
· C	302,09399	1495	310,46021	1.61980	335,50000	4,09208	0.05225100	052251000000.56250	1.4873476	
	713,99307	1.36415	210,26053	1.61089	134 JOORD	4.00177	0.10248100	0.102481000000.56250	1.4897407	
~	325,26147	1.36414	319.07910	1.61900	335.0909E	75066.7	0.15106100	00000.56250	1.4401338	
2	336.24341		319,08380	1.61980	335,00000	C\$080*5	0.10400100	04001000000.56250	. 480570	
0.2	357,19312	1.36417	21 R. RR907	1.61090	134. A9990	0.07747	0.2505510(250551000000,56250	1.4900293	
	376,97778	1.36417	318, 70395	1.41980	134.790AJ	9.07609	0.28186100	281861000000.56250	1.4904804	
2900	395,77515		3] 4, 94462	1.419RO	335,00000	9.07636	0.25624100	256241000000,56250	1.4895850	
~	413.729CR	1.36415	319.650PR	1.41990	33 E. 6990E	0.98254	0.19614100	0.196141000000.56250	1.4864607	
0062	430,96069	.3641	720. 89PK7	1.61980	00000°166	7£ 900° 0	0.12522100	0.125221000000.56250	1.4804086	
006520	447,56250	.3640	222.53709	1.61980	334,69995	10.01741	0.06568100	.085481000000.56250	1.4732254	
0	455,65137	2440	323,36401	1.61080	330 500BE	F.0174A	0.044511000000.54	00000.56250	205044.	
0.2000	463.60864	1.36405	350. 220.05	1.61980	340,50000	ှ	0.020010	0.02001000001100000	1.4453988	
-200	471.44067	6	225.077AA	1.61090	•	5.03012	0.0157110.0	0.015711000000.56750	1,4615173	
0.0	470.15552	1.36401	6.0300	1.41980	0006k° 67k	F.03707	0.0	000000.54750	1.4577268	
				-						

CAMMA	ET AP	NP/107-P3	JP/(PTP-P)	טיין איי	1100/00)	10/10	110/1011
. 3cpco	0.0	0.01242	0.00298	1.0000	1.50822	0000-1	1 16867
00 vo£*1	0.0	0.01200	0.002Ro	1,00000	1,50927	1,0000	1.4603
luose"	0.0	0.01230	0.00277	1.0000	1.50822		1 144.24
20008.1	0.0	0.01346	0.00261	1.00000	1.59822	1,0000	1 16256
-3000E	0.0	0,01291	0.00234	1,00000	1,50822	1,0000	1.16310
£0.00 €	0.0	0.00015	0.00147	1.00000	1.59922	00000	1.16285
£0002.	0.0	0.00230	0.00033	1.00000	1.59822	1.00000	1.16250
₹000E*	0.0	-0.00527	-0.00046	00000	1,5000	1,0000	1.16210
30899	0.0	-0.01024	-0.00118	1.0000	1.59822	1,0000	1 16562
30402	0.0	-0.01111	-0.00116	1.00000	1,50822	00000	1 17014
3880€	0.0	-0.0084a	-0.00082	1.0000	1.50822	00000	1 17606
.39PAO	0.0	-0.00554	-0.00061	1,30000	1.50822	1,0000	1 17017
.39976	0.0	-0.0051k	-0.00046	00000	1.50822	00000	10000
30871	0	-0.00510	-0.00045	1,0000	1.50822	1 00000	1 105.0
.39PF.7	0.0	-0.00710	-0.00059	1.0000	1.59822	1.00000	1.18880

APPENDIX B

GEOMETRIC CONSIDERATIONS FOR SUBSONIC LEADING EDGES ON TRANSONIC ROTOR BLADES

APPENDIX B: GEOMETRIC CONSIDERATIONS FOR SUBSONIC LEADING EDGES ON TRANSONIC ROTOR BLADES

We first note that a simple leading edge configuration is obtained by sweeping each leading edge element dl in the plane formed by the local relative velocity W_1 and the radius (W_1 -r plane). This plane intersects the Mach cone along two generatrices that form the Mach cone angle μ with W. Any other plane through the apex cuts the cone along generatrices forming a smaller angle μ " with W in the W-r projection. Since the radial projection of dl is essentially proportional to $\sin \mu$ ", it follows that the simple case defined above yields the shortest possible swept blade length for a given annulus height and a given relative velocity distribution W(r). For structural reasons, however, the leading edge must be swept aside from the W-r plane.

The general situation is shown in Fig. B.1 (Refer also to Fig. 17).

la shows the projection on a plane perpendicular to the radius passing through leading edge point P. The velocity triangle is projected in that plane for visualization convenience.

lb shows the projection on the W-r plane, with the Mach lines forming the Mach angle μ with W. In general, W forms an angle ϵ_{W_1} with plane la.

lc shows the projection on a plane perpendicular to W, intersecting the Mach cone along circle c.

ld shows the projection on a meridional plane.

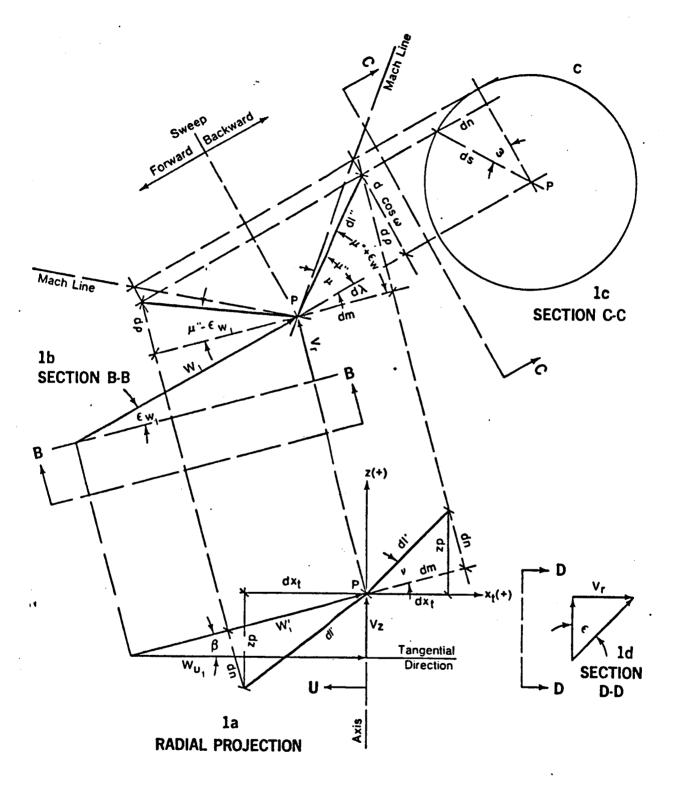


FIG B-1 SONIC SWEPT LEADING EDGE ELEMENT

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u

V

On la, the leading edge element dl has the projection dl' and the radial planes passing through dl and W form the angle ν , which is a design parameter to be selected so as to minimize the blade bending stresses. The resulting lateral sweep component dn appears also on projection lc and causes the Mach angle μ between dl and W to project into the W-r plane lb with a smaller aperture μ ".

From la

$$dx_t = \pm dl' \cos (\beta + \nu)$$

$$dz = \pm dl' \sin (\beta + \nu)$$

In the above and the following relations, the top signs denote backwards, the bottom signs forward sweep.

Since

$$dl' = \frac{dm}{\cos v} = \frac{d\rho}{\tan(\mu'' \pm \varepsilon_{w_{1}}) \cdot \cos v}$$

$$\left[\frac{dx_{t}}{dz}\right] = \frac{\begin{bmatrix}\cos (\beta + v)}{\sin(\mu'' \pm \varepsilon_{w_{1}})} & \frac{1}{\cos v} & d\rho$$

$$\tan(\mu'' \pm \varepsilon_{w_{1}}) & d\rho$$

From 1b and 1c

$$tan \mu'' = \frac{ds \cos \omega}{d\lambda} = tan \mu \cdot \cos \omega$$
 (B.2)

and since

$$\sin \omega = \frac{dn}{ds} = \frac{dn}{dm} \frac{dm}{dl} \frac{dl}{ds} \frac{\cos \omega}{\cos \omega} =$$

=
$$\tan \nu \cdot \cos (\mu'' \pm \epsilon_{W_1}) \frac{\cos \omega}{\sin \mu''}$$

Therefore,

$$\cos \omega = \frac{1}{\sqrt{1 + \frac{\tan^2 v}{\sin^2 u''} \cos^2 (\mu'' \pm \varepsilon_w)}}$$
 (B.3)

which is introduced in Eqn. B.2, yielding

$$tan^{2}\mu'' = \frac{tan^{2}\mu}{1 + \frac{tan^{2}\nu}{\sin^{2}\mu''}} cos^{2} (\mu'' \pm \epsilon_{\mu})$$
(B.4)

Developing cos ($\mu^{\text{"}}$ $\pm \epsilon_{W}^{}$), Eqn. B4 is reduced to a quadratic equation for tan $\mu^{\text{"}}$. The solution is

$$\tan \mu'' = \frac{\pm \sin \epsilon_w \cos \epsilon_w \tan^2 v + \sqrt{\tan^2 \mu (1 + \sin^2 \epsilon_w^{\dagger} \tan^2 v) - \cos^2 \epsilon_w^{\dagger} an^2 v}}{1 + \sin^2 \epsilon_w^{\dagger} \tan^2 v}$$
(B.5)

(Only the (+) sign is valid in front of the radical, since tan μ " must tend toward tan μ when ν tends toward 0).

We define the blade profiles and their stacking in cylindrical coordinates. The angular abcissa of leading edge point P(r) then is

$$\theta_{L}(r) = \theta_{L_{1}} + \int \frac{dx_{t}}{\rho} = \theta_{L_{1}} \pm \int \frac{\cos (\beta + \nu)}{\tan(\mu'' \pm \epsilon_{W_{1}})} \frac{1}{\cos \nu} \frac{d\rho}{\rho} (B.6)$$

$$f_{M=1}$$

$$Z_{L}(r) = Z'_{L_{1}} + \frac{\sin (\beta + \nu)}{\tan (\mu'' \pm \epsilon_{W_{1}} \cos \nu)} d\rho$$

$$r_{M=1}$$
(B.7)

where
$$\varepsilon_{W_1} = \arcsin \frac{V_r}{W_1}$$
 (B.8)

Eqs. (B.6) and (B.7), together with (B.5) and (B.6), determine the cylindrical coordinates of the profile leading edges, for a blade with sonic leading edge.

With the section profile data, the stacking of the centers of gravity is determined, and the blade bending stresses can be calculated. However, it is advisable to iterate the leading edge coordinates until a favorable alignment of the CG's is achieved, prior to the calculation of stresses.

By optimum selection of the lateral sweep and of the radial location of the point of sweep reversal, it is expected that the additional stresses affecting the subsonic leading edge blade will be reduced to:

- (a) Additional centrifugal stresses from the added blade mass necessary to materialize the subsonic leading edge configuration.
- (b) Bending stresses from moments without substantial component in the direction of the axes of minimum inertia.

I

APPENDIX C FUNDAMENTAL ACOUSTICAL ASPECTS OF STATOR DESIGN

APPENDIX C

FUNDAMENTAL ACOUSTICAL ASPECTS OF STATOR DESIGN

C.1 Continuous and Discrete Line Sources in a Stationary Acoustic Medium

Continuous Line Source

Consider a line monopole source of the type

$$i(k_0 x - \omega_0 t)$$

$$q(x,t) = Q_0 e \qquad , \qquad (C.1)$$

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where Q_O is the source strength per unit length. The line source of Eq. C.l represents one wave traveling along the x-axis (see Fig. C.l) with a velocity c_O given by

$$c_{O} = \omega_{O}/k_{O} \tag{C.2}$$

One is interested (only) in the $far\ field$ pressure p(x,y,z,t) radiated by the line source. Define

$$r = (y^2 + z^2)^{\frac{1}{2}}$$
 (C.3)

Consider the case (referred to as Case No. 1) where the source velocity \mathbf{c}_{0} is supersonic, i.e.,

$$|c_0| > c$$
 , (C.4)

or equivalently,

$$|k_0| < k_{a_0} = |\omega_0|/c$$
 . (C.5)

Here c is the sound speed for the medium and k_a is the acoustic wavenumber at frequency ω_O . For this case, the far field pressure p(x,r,t) is non-zero; in other words the line source can radiate acoustic power. More specifically,

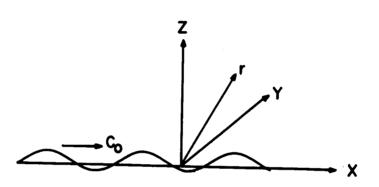


FIG. C.1. SKETCH OF A LINE MONOPOLE SOURCE.

$$p(x,r,t) = constant \times \frac{1}{(k_r r)^{\frac{1}{2}}} e^{i(k_o x + k_r r - \omega_o t)}, \quad (C.6)$$

where the radial wavenumber k_{r} is given by

$$k_r = (k_{a_0}^2 - k_0^2)^{\frac{1}{2}}$$
 (C.7)

Since $k_0^2 < k_{a_0}^2$ (see Eq. C.5), k_r is real and the sound is propagated radially outwards from the x-axis.

Now consider the alternate case (Case No. 2) where the source velocity \mathbf{c}_{O} is subsonic, i.e.,

$$|c_0| < c$$
 , (c.8)

or equivalently,

$$|k_0| > k_{a_0}.$$
 (C.9)

For this case, the far field pressure p(x,r,t) is zero; in other words the line source cannot deliver any acoustic power. More specifically,

$$p(x,r,t,) = 0$$
 (C.10)

. ...

This happens because the radial wavenumber $k_{\mathbf{r}}$ is imaginary,

$$k_r = +i (k_0^2 - k_{a_0}^2)^{\frac{1}{2}}$$
, (C.11)

and the near field pressure decays exponentially in the radial direction.

Let us reconsider the above results in terms of the spatial Fourier transform $\tilde{q}(k_1,t)$ of Eq. (C.1). First, define the general Fourier transforms.

$$\tilde{q}(k_1,t) = \frac{1}{2\pi} \int q(x,t) e^{-ik_1 x} dx$$
 (C.12)

$$q(x,t) = \int \tilde{q}(k_1,t) e^{ik_1x} dk_1$$
 (C.13)

Unless stated otherwise, the limits of integration are always to be taken from $-\infty$ to $+\infty$. For later use, the temporal Fourier transforms shall also be needed, defined as follows.

$$\tilde{q}(k_1, \omega) = \frac{1}{2\pi} \int \tilde{q}(k_1, t) e^{i\omega t} dt$$
 (C.14)

$$= \frac{1}{(2\pi)^2} \int \int q(x,t) e^{-i(k_1 x - \omega t)} dxdt \qquad (C.15)$$

$$q(x,t) = \int \int \tilde{q}(k_1, \omega) e^{i(k_1 x - \omega t)} dk_1 d\omega \qquad (C.16)$$

$$= \int \tilde{q}(x,\omega) e^{i\omega t} d\omega \qquad (C.17)$$

Substituting q(x,t) of Eq. (C.1) into Eq. (C.12),

$$\tilde{q}(k_1,t) = Q_0 e^{-1\omega_0 t} \delta(k_1 - k_0), \qquad (C.18)$$

where δ is the Dirac delta function. Figure C.2 illustrates $\tilde{q}(k_1,t)$ for Case No. 1 (radiation) and for Case No. 2 (no radiation). The "radiation span" along the wavenumber k_1 is centered around the wavenumber k_1 = 0 and ranges from -k_a to +k_a. This radiation span is shown in Fig. C.2 as a $^{\circ}$ shaded strip.

Let us reformulate the condition of no radiation in terms of velocities and Mach numbers. The two extremes $-k_a$ and $+k_a$ of the radiation span correspond respectively to the olowest and the highest velocities that the source wave can have

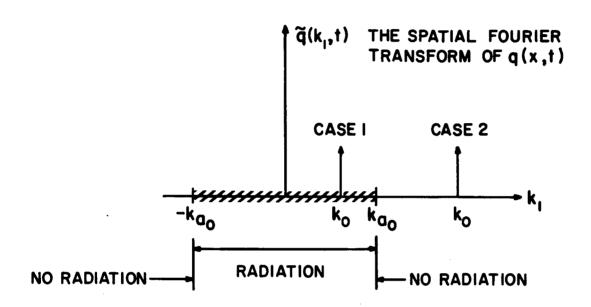


FIG. C.2. CASES OF RADIATION (No. 1) AND NO RADIATION (No. 2) ILLUSTRATED IN TERMS OF THE WAVENUMBER $k_{_{\parallel}}$.

for no radiation to occur (Case 2). These extreme velocities and Mach numbers are,

$$c_{\ell} = \frac{\omega_{O}}{-k_{a_{O}}} = -c$$

$$m_{\ell} = \frac{c_{\ell}}{c} = -1$$

$$c_{u} = \frac{\omega_{O}}{+k_{a_{O}}} = +c$$

$$m_{u} = \frac{c_{u}}{c} = +1$$
(C.19)

The source wave Mach number m_0 is defined as:

$$m_{o} = \frac{c_{o}}{c} \tag{C.21}$$

Thus, the condition of no radiation (Case 2) becomes

$$m_{\ell} < m_{o} < m_{u}$$
 (C.22)

Next, consider a spatially frozen but otherwise arbitrary pattern q(x,t) traveling, as before, with fixed velocity c_0 in the x-direction. Thus,

$$q(x,t) = Q(x-c_0t)$$
 (C.23)

In contrast to Eq. (C.1), for which there was one wavenumber k_0 , one frequency ω_0 , and one velocity c_0 , for Eq. (C.23) there is a range of wavenumbers k_1 , a corresponding range of frequencies ω , and one velocity c_0 . Using Eq. (C.15), the Fourier transform of q(x,t) of Eq. (C.23) is

$$\tilde{q}(k_1, \omega) = \frac{1}{(2\pi)^2} \int \int Q(x-c_0t) e^{-i(k_1x-\omega t)} dxdt$$
 (C.24)

$$= \frac{1}{2\pi} \int \tilde{Q}(k) e^{-ik_1 c_0 t + i\omega t} dt \qquad (C.25)$$

$$= \tilde{Q}(k) \delta(\omega - k_1 c_0) , \qquad (C.26)$$

where

$$\tilde{Q}(k) = \frac{1}{2\pi} \int Q(x) e^{-ik_1 x} dx \qquad (C.27)$$

Figure C.3 shows the straight lines along which $\tilde{q}(k_1,\omega)$ of Eq. (C.26) is non-zero. Analogous to Fig. C.2, straight lines corresponding to Case 1 (radiation) and Case 2 (no radiation) are illustrated. Notice that as frequency ω increases, the radiation span $2k_a$ over wavenumber k_1 also increases linearly, But, as long as the source velocity magnitude $|c_0|$ is subsonic, there is no radiation to the far field. Incidentially, the upper right and the lower left quadrants of ω - k_1 plane correspond to positive phase velocities, (i.e., velocities along increasing x), whereas the upper left and the lower right quadrants of ω - k_1 plane correspond to negative phase velocities.

This completes the discussion of a continuous frozen pattern of line sources in a stationary acoustic medium. Consideration of the fact that the acoustic medium is moving uniformly, will simply alter the radiation span along k_1 , as will be discussed in Sec. C.2. However, a frozen convecting pattern along x_1 , will or will not radiate, by exactly analogous rules as developed here, i.e., in terms of the convection or phase velocity c_0 of the pattern.

Finally, note that for a continuous convecting line source of *finite* length, even if the convection velocity co is subsonic, there will be inevitable radiation from the two ends of the line source. For low enough frequencies, the two ends may be less than half an acoustic wavelength apart, in which case there may

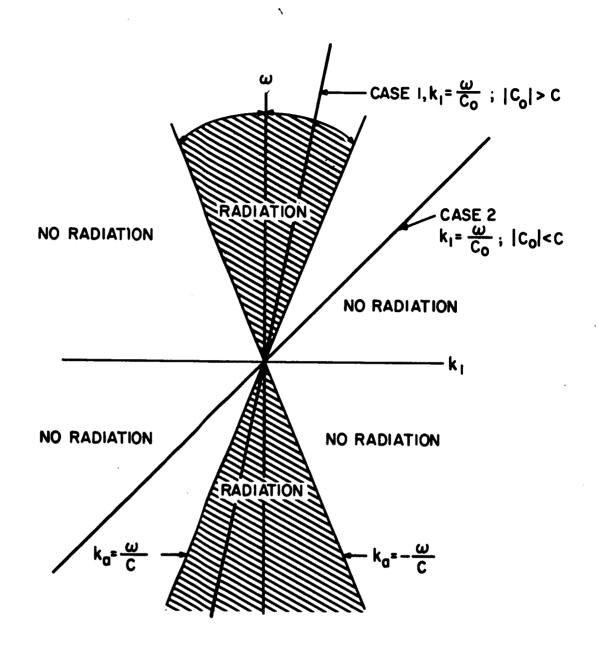


FIG. C.3. CASES OF RADIATION AND NO RADIATION FOR A SPATIALLY FROZEN ARBITRARY PATTERN, ILLUSTRATED IN THE $_{\mbox{\tiny 1}}$, ω PLANE.

be partial cancellation from the two end sources. For higher frequencies, the two end sources will radiate independently. This last remark is discussed more fully below when discrete line sources are considered in a stationary acoustic medium. However, the SBLE is regarded as a continuous line array, and a typical rotor wake impinging on it has a *local* convection velocity co along the *span* of the SBLE. Thus the preceding discussion of continuous line sources, or rather its related extension in Sec. C.2, where account is taken also of the moving-medium acoustics, is applied to determine the SBLE sweep; the criterion that is applied is in terms of the spanwise local velocity of the rotor wake along the SBLE.

Discrete Line Source

Now consider an array of equispaced coherent monopoles, spaced d apart (see Fig. C.4), where:

$$x_j = d_j$$
 , $j = 0, \pm 1, \pm 2 \dots$ (C.28)

In analogy with Eq. (C.1), consider one wavenumber k_0 , one frequency ω_0 and the corresponding phase velocity c_0 . Thus, the source number j has the strength $q(x_j,t)$ given by

$$q(x_j,t) = Q_0 e^{i(k_0 x_j - \omega_0 t)} \delta(x-d_j)$$
 (C.29)

The entire source strength can be written as

$$q(x,t) = Q_0 \sum_{j=-\infty}^{+\infty} e^{i(k_0 x - \omega_0 t)} \delta(x-dj) , \qquad (C.30)$$

and the phase velocity c_{0} , as before, is given by

$$c_0 = \omega_0/k_0 \quad . \tag{C.31}$$

Eq. (C.12) is used to find the spatial Fourier transform of $q(\mathbf{x},t)$ of Eq. (C.30),

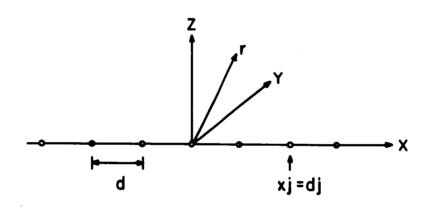


FIG. C.4. SKETCH OF AN ARRAY OF POINT SOURCES.

$$\tilde{q}(k_1,t) = \frac{Q_0}{2\pi} \int e^{-ik_1 x} e^{i(k_0 x - \omega_0 t)} \sum_{j=-\infty}^{+\infty} \delta(x-dj) dx$$
(C.32)

$$= \frac{Q_0}{2\pi} e^{-i\omega_0 t} \sum_{j=-\infty}^{+\infty} e^{-i(k_1 - k_0)dj}$$
 (C.33)

$$= \frac{Q_0}{d} \quad e^{-i\omega_0 t} \sum_{m=-\infty}^{+\infty} \delta(k_0 - k_1 - \frac{2\pi m}{d})$$
 (C.34)

 $\tilde{q}(k_1,t)$ thus consists of an infinite string of Dirac delta functions equispaced along the wavenumber k_1 , the spacing between two adjacent delta functions being $2\pi/d$. It is only the "fundamental mode" or harmonic at $k_1=k_0$ (for m=0 in Eq. C.34) that corresponds to the trace velocity c_0 of Eq. (C.31). The remaining infinite harmonics correspond to infinite other velocities. The rule of radiation (Case 1) or no radiation (Case 2) is exactly the same as the one developed for the continuous array and illustrated in Fig. (C.2). If the fundamental mode or any harmonic(s) lie within the radiation span (- k_0 , k_0), radiation will occur from the fundamental mode or from the harmonic(s) lying within the radiation span.

However, a more interesting and new feature of the discrete array is the classification based on a different criterion. That classification is as follows:

Case A,
$$\frac{2\pi}{d} > 2k_{a_0}$$
, or $d < \frac{\lambda_{a_0}}{2}$ (C.35)

Case B,
$$\frac{2\pi}{d} < 2k_{a_0}$$
, or $d > \frac{\lambda_{a_0}}{2}$ (C.36)

For Case A, the spacing $2\pi/d$ along wavenumber k, between harmonics is greater than the radiation span $2k_{a_0}$, since $k_{a_0} = 2\pi/\lambda_a$, where $\lambda^{\iota}_{a_0}$ is the acoustic wavelength at frequency ω , o o the same condition is expressed by the statement that the array spacing d is smaller than half the acoustic wavelength. For

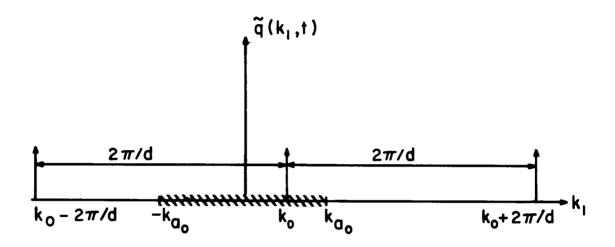
Case B, the opposite situations occur in the wavenumber and spatial descriptions.

The importance of these two cases is depicted in the next few figures. Figure C.5 describes Case Al (the numbers 1 and 2 denote the older classification, 1 corresponds to radiation occurring, 2 corresponds to radiation not occurring). The radiation occurs from the fundamental mode at $k_1=k_0$, but since $2\pi/d>2k_a$, no other harmonic can radiate. Figure C.6 also describes Case Al, however, this time a harmonic, and only one harmonic radiates. Figure C.7 shows the Case A2, a situation one would hope to achieve. The fundamental mode at $k_1=k_0$ lies just outside the radiation span, and no harmonic lies within the radiation span, hence no radiation occurs. Note that for this desired situation, the constraint of Eq. (C.8) (or equivalently of Eq. C.22) as well as the constraint of Eq. (C.35) applies.

Finally, Fig. C.8 shows Case Bl. There is no Case B2. Radiation must occur through same mode(s), whether the phase velocity co is subsonic or supersonic. In other words, arranging for Case B, i.e., having array spacing d greater than half the wavelength, is basically a poor design.

Note that in contrast to the continuous line array for which the discussion related to Fig. C.2 for frequency $\omega_{\rm O}$ could be generalized to discussion related to Fig. C.3 for all frequencies, the discussion of the discrete array presented above cannot be similarly generalized to all frequencies. This is because the array spacing d is in general fixed, whereas the radiation span $2k_{\rm a}$ increases linearly with increasing $\omega_{\rm O}$. Thus, Case A for $^{\rm O}$ frequency $\omega_{\rm O}$ is bound to merge into Case B at some higher frequency.

Aside from the re-definition in Sec. C.2 of the radiation span in wavenumber k_1 , induced by consideration of moving-medium acoustics, the above discussion of a discrete array is applied to determine the number of stator blades, the spacing d corresponding to the circumferential spacing between two adjacent stator tips, and frequency $\omega_{\rm O}$ corresponding to the blade passage frequency.



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FIG. C.5. CASE AT FOR A DISCRETE ARRAY; RADIATION FROM THE FUNDAMENTAL HARMONIC AT \mathbf{k}_{o} .

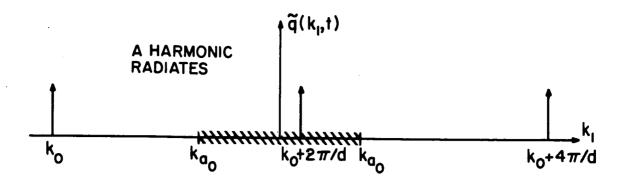


FIG. C.6. CASE A1; RADIATION FROM A HARMONIC OTHER THAN THE FUNDAMENTAL.

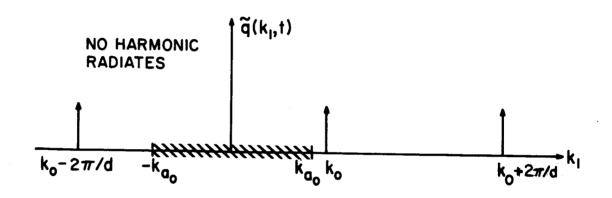
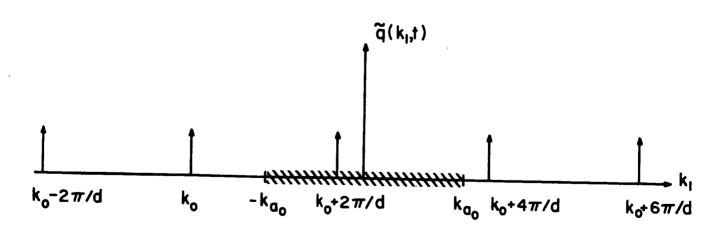


FIG. C.7. CASE A2; NO RADIATION.



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FIG. C.8. CASE B1; INEVITABLE RADIATION.

C.2 Acoustics of a Moving Medium

The only task that needs to be performed in this section is to investigate how the radiation span $(-k_a, +k_a)$ along the wavenumber k_1 gets modified due to the ofact that the acoustic medium is moving uniformly with subsonic velocity $\underline{u} = (u_1, u_2, u_3)$, where u_1, u_2, u_3 are the velocity components in the x, y and z directions.

Once again, consider the line monopole source of Fig. C.1, with Eqs. (C.1) through (C.5) still applicable. In addition to the radial coordinate r of Eq. (C.3), the corresponding radial vector r is defined as

$$r = (y,z) \tag{C.37}$$

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The far field pressure P(x,r,t) now must satisfy the following field equation, (Morse and Ingard, 1964),

$$\nabla_{\mathbf{p}}^{2} + k_{\mathbf{a}_{0}}^{2} \left[1 + \frac{\mathbf{i}}{\omega_{0}} \left(u_{1} \frac{\partial}{\partial \mathbf{x}} + u_{2} \frac{\partial}{\partial \mathbf{y}} + u_{3} \frac{\partial}{\partial \mathbf{z}} \right) \right]^{2} \mathbf{p} = 0 \quad (C.38)$$

For $u_1 = u_2 = u_3 = 0$, Eq. (C.38), reduces to the usual Helmholtz equation applicable for a stationary acoustic medium. In analogy with Eq. (C.11), a criterion is needed that the radial wavenumber k_r must satisfy for no radiation to occur to the far field (i.e., Case 2). However, in analogy with generalization of Eq. (C.3) to Eq. (C.37), a radial wavenumber vector k_r is defined as

$$\underline{\mathbf{k}_{\mathbf{r}}} = (\mathbf{k}_{2}, \mathbf{k}_{3}) , \qquad (C.39)$$

where k_2 and k_3 are the wavenumber components of the radially outwards propagating wave.

Now the important phase aspect of the far field pressure $P(x,\underline{r},t)$ (for a given value of \underline{r} in the farfield) is given by the correspondingly generalized version of Eq. (C.6)

$$P(x,\underline{r},t) = constant \times e^{i(k_0x - \frac{k_r}{r} \cdot \underline{r} - \omega_0t)}, \qquad (C.40)$$

where

$$\underline{\mathbf{k}}_{\underline{\mathbf{r}}} \cdot \underline{\mathbf{r}} = \mathbf{k}_{2} \mathbf{y} + \mathbf{k}_{3} \mathbf{z} \tag{C.41}$$

The following definitions are required:

$$k_r = |k_r| = (k_2^2 + k_3^2)^{1/2}$$
 (C.42)

$$u_r = |u_r| = (u_2^2 + u_3^2)^{1/2}$$
 (C.43)

$$k_2 = k_r \sin \alpha_k$$
, (C.44)

$$k_{3} = k_{r} \cos \alpha_{k} , \qquad (C.45)$$

$$u_2 = u_r \sin \alpha_u \quad , \tag{C.46}$$

$$u_{3} = u_{r} \cos \alpha_{u} , \qquad (C.47)$$

so that

$$\frac{k_r}{r} \cdot u_r = k_2 u_2 + k_3 u_3 = k_r u_r \cos \alpha , \qquad (C.48)$$

where

$$\alpha = (\alpha_k - \alpha_{ij}) . \qquad (C.49)$$

Thus, α is the angle between the radial wavenumber vector $\mathbf{k_r}$ (or the radial location vector (y,z) of observation in the farfield) and the radial flow vector $\underline{\mathbf{u_r}}$.

Now, substituting Eq. (C.40) into Eq. (C.38) gives the following relation,

 $-k_{r}^{2} - k_{o}^{2} + k_{a_{o}}^{2} \left[1 - \frac{1}{\omega_{o}} \left(u_{1}k_{o} + u_{r}k_{r}\cos\alpha\right)\right]^{2} = 0$ (C.50)

which can be rewritten as a quadratic equation in k_{r} as follows

$$Ak_{r}^{2} + Bk_{r} + C = 0$$
 (C.51)

where

$$A = \frac{k_a^2}{\omega_0^2} u_r^2 \cos^2 \alpha - 1 = m_r^2 \cos^2 \alpha - 1$$
 (C.52)

$$B = 2 \frac{k_{a_{o}}^{2}}{\omega_{o}^{2}} u_{1}k_{o}u_{r} \cos \alpha - 2 \frac{k_{a_{o}}^{2}}{\omega_{o}} u_{r} \cos \alpha$$

$$= 2(m_1 m_r k_0 \cos \alpha - k_{a_0} m_r \cos \alpha)$$
 (C.53)

and

$$C = (-k_0^2 + k_{a_0}^2 - 2 \frac{k_{a_0}^2}{\omega_0} u_1 k_0 + \frac{k_{a_0}^2 u_1^2}{\omega_0^2} k_0^2)$$

$$= (-k_0^2 + k_{a_0}^2 - 2k_{a_0}^m k_0 + m_1^2 k_0^2)$$
 (C.54)

In the above equations appropriate Mach numbers are introduced by division of velocities by the sound speed c = ω_0/k_{a_0} .

For the stationary acoustic medium, the condition on radial wavenumber magnitude k_r , for no radiation to occur (Case 2), was that k_r be imaginary (see Eq. C.ll). In analogy with that requirement for no radiation to occur, it is required that k_r of Eq. (C.51) be complex (with positive imaginary part). That will happen if and only if

$$AC - B^2/4 > 0$$
 . (C.55)

Now, the left hand side of Eq. (C.55) does not contain k_r , but is a quadratic form in k_o , the wavenumber of the source wave. Thus, Eq. (C.55) can be rewritten as

$$DK_{O}^{2} + Ek_{O} + F > 0$$
 , (C.56)

where

$$D = 1 - m_1^2 - m_{r'}^2 \cos^2 \alpha \qquad (C.57)$$

$$E = 2k_{a_0} m_1 \qquad (C.58)$$

$$F = -k_{a_0}^2 \tag{C.59}$$

The minimum value of D occurs for α = 0 or π (i.e., when $\frac{k_{r}}{m}$ and $\frac{k_{r}}{m}$ are coincident or oppositely directed. This minimum walue D_{minm} is given by

$$D_{minm} = 1 - m_1^2 - m_r^2 = 1 - m^2 > 0$$
,

where m is total flow Mach number. Since D_{minm} , and therefore D is always positive, the left hand side of Eq. (C.56) is positive for large $|k_0|$ (i.e., for $k_0 + \pm \infty$), being dominated by the first term Dk_0^2 . This behavior, incidently, is consistent with the inequality expressed by Eq. (C.56). Recalling that the inequality of Eq. (C.56) is the condition on k_0 for no radiation to occur (i.e., Case 2), the radiation will, in fact, take place for a range of wavenumbers k_0 of relatively small magnitude. This range, the radiation span along axial wavenumber k_1 , is determined by the two real roots k_1 and k_2 of the quadratic form of Eq. (C.56)

$$k_{a_{0\pm}} = k_{a_{0}} \frac{-m_{1}^{\pm}(1-m_{r}^{2}\cos^{2}\alpha)^{\frac{1}{2}}}{(1-m_{1}^{2}-m_{r}^{2}\cos^{2}\alpha)}$$
 (C.60)

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In analogy with Fig. C.2, this radiation span is shown as a shaded strip in Fig. C.9. The center O' of the span is shifted to the left by the amount k_a m₁/D (with D given by Eq. (C.57) from the origin O ($k_1=0$). O This shift resulting from first (common) term on the right hand side of Eq. (C.60), is interpreted as a Galilean shift. The equal intervals (k_a , O') and O', k_a), resulting from the second terms on the right o- hand side of Eq. (C.60) are interpreted as Lorentz half-spans.

Also shown in Fig. C.9 is $\tilde{q}(k_1,t)$ for a phase wave whose phase velocity c_0 is supersonic in the fixed coordinate system (or equivalently whose k_0 is less than k_{a_0}), yet since k_0 lies outside the radiation span, the particular phase wave illustrated will not radiate to the farfield.

Reformulation of the condition of no radiation in terms of Mach numbers can be done along exactly similar lines as done in Eqs. (C.19), (C.20) and (C.21). Thus, the upper and lower permissible Mach numbers $\mathbf{m}_{\mathbf{u}}$ and $\mathbf{m}_{\mathbf{l}}$ are given by

$$m_{\ell} = \frac{(1-m_{1}^{2}-m_{r}^{2}\cos^{2}\alpha)}{-m_{1}-(1-m_{r}^{2}\cos^{2}\alpha)^{\frac{1}{2}}} = m_{1}-(1-m_{r}^{2}\cos^{2}\alpha)^{\frac{1}{2}}, \quad (C.61)$$

$$m_{u} = \frac{(1-m_{1}^{2}-m_{r}^{2}\cos^{2}\alpha)}{-m_{1}+(1-m_{r}^{2}\cos^{2}\alpha)^{\frac{1}{2}}} = m_{1}+(1-m_{r}^{2}\cos^{2}\alpha)^{\frac{1}{2}}, \quad (C.62)$$

and for no radiation to occur, the following must be satisfied:

$$m_{\ell} < m_{O} < m_{u}$$
 (C.63)

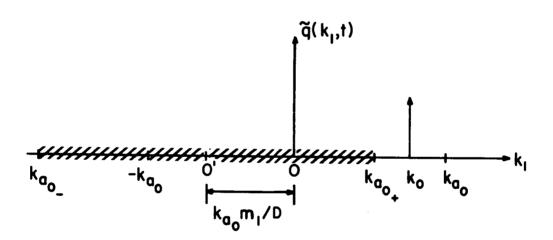


FIG. C.9. SKETCH OF RADIATION SPAN ALONG WAVENUMBER $k_{_{\scriptsize{1}}}$ FOR A MOVING ACOUSTIC MEDIUM.

Extension similar to that from Figs. C.2 to C.3 can also be easily performed for the present case; as a result of the medium motion, the acoustic "cones" of radiation in the $\omega-k_1$ plane will be asymmetrical about the ω axis.

Finally, the entire discussion of discrete arrays in Sec. C.2 can be applied here with the newly defined radiation span.

C.3 An Estimate Of Overall Power Radiated From The Stator

Figure C.10 shows the wake velocity deficits as viewed in time at one SBLE near the tip. There are f_r such deficits per second, where f_r = 8600 Hz is the rotor blade passage frequency. Each individual wake deficit, v(t) has an approximately Gaussian shape around its peak deficit value v_0 , thus

$$v(t) \approx v_0 e^{-t^2/2T^2}$$
 (C.64)

where the "standard deviation" T, and the maximum deficit \mathbf{v}_{o} are estimated to be:

$$T \approx 9.6 \times 10^{-6} \text{ sec.}$$
 (C.65)

$$v_o \approx 44 \text{ m/sec (144 ft/sec)}$$
 (C.66)

The maximum deficit, v corresponds to $10^{\rm O}$ change in angle of attack. The time interval τ between consecutive deficits is given by

$$\tau = \frac{1}{f_r} \approx 1.16 \times 10^{-4} \text{ sec.}$$
 (C.67)

Note that τ is about an order of magnitude greater than T, in other words the wake deficits are narrow in time when compared to their rate of arrival.

The above data regarding the wake velocity deficits was developed from Kemp and Sears (Ref. 18). The description of the wake velocity deficit in a spatial coordinate, x, can be obtained by assuming that the wakes arrive at the SBLE tips as (locally) frozen spatial patterns, being convected along with the local gas speed V, where

$$V = (m_c^2 + m_a^2)^{1/2} c \approx 195 \text{ m/s } (641 \text{ ft/sec})$$
 (C.68)

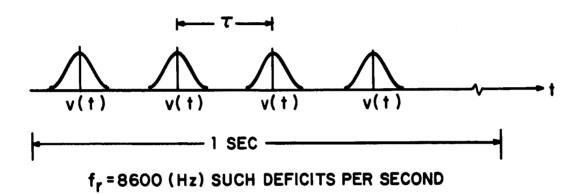


FIG. C.10. SKETCH OF TIME HISTORY OF WAKE VELOCITY DEFICITS AS THEY IMPINGE ON A SINGLE SBLE TIP.

 $\rm m_{\rm c}$ and $\rm m_{\rm a}$ being given by Eqs. 2,1 and 22 . Thus, the spatial picture of a wake velocity deficit is obtained by the transformation,

$$x = Ut . (C.69)$$

For estimating the overall acoustic power radiated from the stator it is convenient, as shown below, to integrate the results in the time domain. However, in order to get a qualitative understanding of the situation in the frequency domain we discuss briefly the Fourier transform $\tilde{v}(\omega)$ of v(t) of Eq. (C.64).

$$\tilde{v}(\omega) = \frac{1}{2\pi} \int v(t)e^{i\omega t} dt$$
 (C.70)

$$= \frac{\mathrm{Tv}_{0}}{(2\pi)^{1/2}} e^{-\omega^{2}\mathrm{T}^{2}/2}$$
 (C.71)

The Fourier transform $\tilde{v}^{\, \prime}(\omega)$ of the sequence of pulses of Fig. C.10 may then be written as

$$\tilde{v}'(\omega) = \frac{1}{2\pi} \int v'(t)e^{i\omega t} dt$$

$$= \frac{1}{2\pi} \int \sum_{j=-\infty}^{+\infty} v(t-\tau,j) e^{i\omega t} dt \qquad (C.72)$$

$$= \tilde{v}(\omega) \omega_{r} \sum_{n=-\infty}^{+\infty} \delta(\omega-n\omega_{r}) , \qquad (C.73)$$

where ω_{0} is the blade passage frequency in radians/sec (see Eqs. 0 17 and 18),

$$\omega_{O} = 2\pi f_{r} = \frac{2\pi}{\tau} = \Omega B . \qquad (C.74)$$

Thus, as expected, the frequency content of the rotor wake velocity deficits v'(t) consists of the various rotor harmonics n. Since $\tilde{v}(\omega)$ does not decay appreciably with increasing frequency (the "standard deviation" of $\tilde{v}(\omega)$ is 1/T) the higher rotor harmonics at $n = \pm 2$, ± 3 , etc. have nearly the same strength or amplitude as the fundamental harmonic at $n = \pm 1$.

Now, reverting back to the time domain analysis, the fluctuating lift $\ell(t)$ generated at the leading edge of a SBLE tip from impingement of one wake deficit v(t) is given by

$$\ell(t) = \int_{0}^{\infty} v(t') h(t-t') dt', \qquad (C.75)$$

where h(t) is the unit impulse response function derived from Küssner's function [Bisplinghoff et al., Ref. 19]. Since the essential uncancelled fluctuating lift is restricted to a relatively small segment of the SBLE span near the tip, use of Küssner's function, valid for low aspect ratio, is readily justified for the present calculation. Since Küssner's function gives the lift due to a sharp-edged gust (i.e., due to a gust that is a unit step function), the unit impulse response function h(t) is obtained by differentiating the Küssner's function. h(t), so obtained, is given by

$$h(t) = C_{L} \left[\frac{0.13}{2\tau_{b}} e^{-0.13t/\tau_{b}} + \frac{1}{2\tau_{b}} e^{-t/\tau_{b}} \right],$$
 (C.76)

where τ_{b} is the time taken by the gust to travel the (swept) semichord b.

$$\tau_{\rm b} = b/U \approx 1.33 \text{ x } 10^{-4} \text{ sec.}$$
 (C.77)

The above estimate of τ_b is based on b ≈ 0.034 m (0.11 ft), and U of Eq. C.68. The lift coefficient C_T is given by

$$C_L \approx 2\pi\rho Ub \frac{\lambda_r}{4}$$
, (C.78)

where ρ is the medium density (2.4×10^{-3} lb-sec²/ft⁴ ≈ 1.24 Kg/m³) and λ_r is the acoustic wavelength at blade passage frequency f_r (Eq. 26). Note that in Eq. C.78, $\lambda_r/4$ denotes a rough estimate of the SBLE span near the tip from which the uncancelled fluctuating lift is estimated to radiate. Now, this choice of $\lambda_r/4$ is suitable (only) for the rotor fundamental harmonic at frequency ω_r . For the higher rotor harmonics of Eq. C.73, correspondingly smaller spanwise length scales would be more appropriate. However, in the time domain analysis that is being pursued, the choice of $\lambda_r/4$ in Eq. C.78 is taken to apply to all the rotor harmonics, therefore the resulting estimate of the overall (i.e., frequency-integrated) radiated power is liable to be conservative.

Substituting Eqs. (C.64) and (C.76) into Eq. (C.75), enables calculation of fluctuating lift $\ell(t)$ at a single SBLE tip due to the impingement of a single wake velocity deficit v(t). Since the minimum time constant τ_b of h(t) is much larger than the time constant or "standard" deviation" T of v(t) [compare Eqs. (C.65) and (C.77)], for evaluating $\ell(t)$ from Eq. (C.75), one can justifiably approximate v(t) of Eq. (C.64) as follows.

$$v(t) \approx v_0^{(2\pi)^{1/2}} T\delta(t)$$
 (C.79)

The constant $(2\pi)^{1/2}$ T in Eq. (C.79) is introduced so as to make the total "area" (in other words, the integral $\int v(t) dt$ the same for Eqs. (C.64) and (C.79). From Eq. (C.70) and (C.71), note that this area is equal to $2\pi\tilde{v}(\omega)_{|\omega=0}$. Substituting Eq. (C.79) into Eq. (C.75), gives

$$\ell(t) = v_0 (2\pi)^{1/2} T h(t)$$
 (C.80)

From the point of view of generation of steady lift, the stator blade chord is expected to be oriented parallel to the flow velocity U at its leading edge, so that a zero mean angle of attack is ensured. Hence, the wake-deficit-induced fluctuating lift of Eq. C.80 is oriented normal to the flow velocity U. The acoustic intensity I(t) radiated by this "transverse" dipole (i.e., the direction of fluctuating lift is normal to flow) is given by [Lighthill, Ref. 15; Morse and Ingard, Ref. 17],

$$I(t) = [2l(t)]^2 \frac{1}{12\rho\pi c^3} G_2(m) , \qquad (C.81)$$

where l(t) = d/dt l(t) and $G_2(m)$ is a function of the flow Mach number m = U/c,

$$G_2(m) = \frac{3}{4} \left[\frac{2}{m^2(1-m^2)} - \frac{1}{m^3} \ln \frac{1+m}{1-m} \right].$$
 (c.82)

The factor 2 appearing with $\ell(t)$ in Eq. C.81 accounts for the baffling effect due to the presence of the duct wall (assumed to be acoustically rigid) enclosing the SBLE tip.

The radiated acoustic energy E, associated with the intensity I(t) of Eq. C.81 is given by

$$E = \int_{0}^{\infty} I(t) dt. \qquad (C.83)$$

From Eqs.C .76, C.80 and C.81, we note that the only time dependent factor of I(t) involves h(t) of Eq. C.76, hence, the integral to be evaluated is

$$\int_{0}^{\infty} \dot{h}(t)^{2} dt = C_{L}^{2} \frac{0.133}{\tau_{b}^{3}} , \qquad (C.84)$$

thus, substituting Eqs. C.80, C.81 and C.84 into Eq. C.83 we get

$$E = \frac{2}{3} \frac{G_2(m)}{\rho c^3} (v_0 TC_L)^2 \frac{0.133}{\tau_b^3}$$
 (C.85)

Now, E is the acoustic energy radiated from a single SBLE tip due to impingement of a single wake velocity deficit v(t). Hence, the acoustic power radiated from a single SBLE tip is Ef_r , where f_r is the rate of impingement of wake deficits on the SBLE tip.

Next, assume that the V individual SBLE tips radiate more or less incoherently (an assumption particularly valid for higher rotor harmonics n of Eq. (0.73)). Hence, the power radiated from the V stator tip sources is (0.73)).

}

Finally, even though the calculation for the power radiated from the stator leading edge sources at the hub is not carried out separately, because of closer circumferential spacing between these hub sources, the total power radiated from the stator hub is likely to be considerably less than that from the stator tip. The total power II radiated from the stator is conservatively estimated to be given by

$$\Pi = 2Ef_{\mathbf{r}}V \tag{C.86}$$

Substituting in Eqs. (C.85) and (C.86) the numerical values quoted for various parameters (with V=59) gives

$$\Pi = 130.5 \text{ dB re } 10^{-12} \text{ watt}$$
 (C.87)

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APPENDIX D NOTES ON EMPIRICAL CALCULATION OF FAN NOISE LEVELS

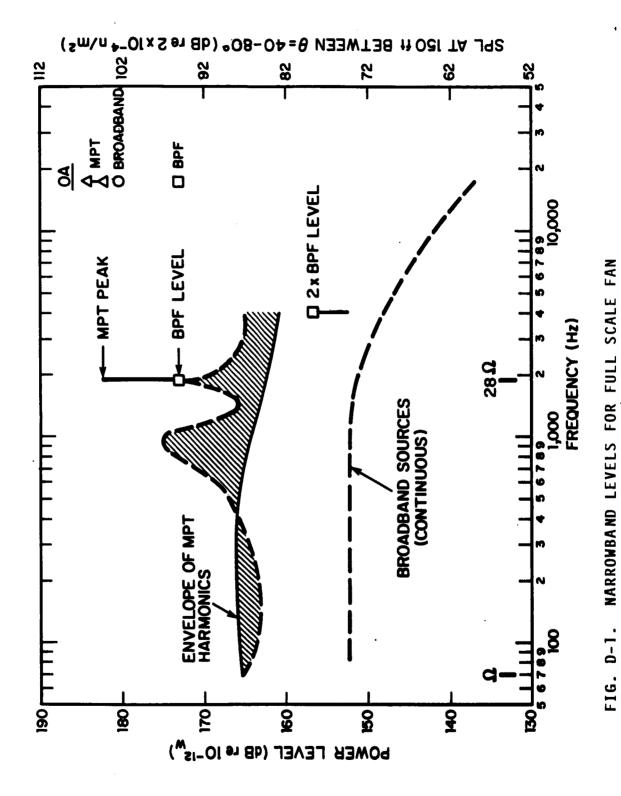
APPENDIX D: NOTES ON EMPIRICAL CALCULATION OF FAN NOISE LEVELS

As mentioned in Section 7, all components of the rotor and stator noise spectrum could not be calculated from basic considerations. The previous Appendix gives a noise level calculation for the residual stator noise sources. In the interest of determining what reduction in levels the swept rotor might be expected to cause, Burdsall's empirical correlation (Ref. 19), was exercised for both the actual fan model and a "full scale" counterpart. The parameters required in Burdsall's routine are given in Table D-1. Figure D-1 summarizes the narrow band power levels and spectra for the various components. (SPL arbitrarily computed at 150 ft., 60° from rotor axis) showing the predominance of MPT'S. Of course, the details of the MPT spectrum vary from fan to fan due to their origin in manufacturing tolerances. Fig. D-2 shows a typical comparison of Burdsall's prediction with measured data, indicating a fairly large fluctuation in harmonic levels around the mean line of the prediction. In Fig. D-1, it is clear that according to this scheme, elimination of MPT's would reduce the tone levels considerably. However, note that in Fig. D-1, the line is an envelope of discrete frequency levels at various multiples of rotation rate while the broadband spectrum is continous. Thus, integration into constant percentage bandwidths, and into overall levels will lead to the MPT and broadband levels being very nearly identical.

As a final point, it is interesting to note that the power level of the BPF tone (non-MPT noise) is ~10 dB above the predicted level for the swept stator as described in Appendix C.

INPUT PARAMETERS FOR EMPIRICAL NOISE PREDICTION TABLE D-1

SYMBOL	DESCRIPTION	UNITS	MODEL FAN	FULL SCALE	BROADBAND	DISCRETE	COMBINATION
DIAT	fan tip diameter	inches	20.0	. 89.0	×	×	×
DIAH	fan hub diameter	inches	9.5	42.0		1	: ×
Ø	number of blades	-	28.0	28.0			: ×
NV	number of vanes	-	39.0	59.0			
BPR	bypass ratio	. }	8.0	8.0	×		
GAP	rotor/stator space	inches	3.94	27.6			
CHORD	blade tip chord	Inches	3.15	14.0		-	
TXGGG	fan tip dia. gradient	1	-0.22	-0.222			*
нхааа .	fan hub dia. gradient	1 1	777.0	11110			×
RS	blade suction surface radius of curvature	inches	12.0 (avg.)	54.0			*
SIGPHI = 0	standard deviation of blade tip metal angle	radians	0.015	0.015			×
חנ	inlet duct length	feet	1.5	7.9		×	
FFR	radius to observer	feet	50.0	150.0	×	×	
RPM	fan speed	rpm	18450	4130.0			
SPAF	specific airflow rate	1b/sec/ft2	32.8	32.0			
XMXTIP	tip axial Mach number		0.62	0.62			×
P.R.	fan pressure ratio		1.6	1.6		×	
DF	diffusion factor (avg)	1	· · • • • • • • • • • • • • • • • • • •	7.0	×		
TT2	ambient inlet temp- temperature	Н	94	46.0			
PT2	ambient inlet pressure	in. Hg	29.92	29.92			• •
				J	¥		

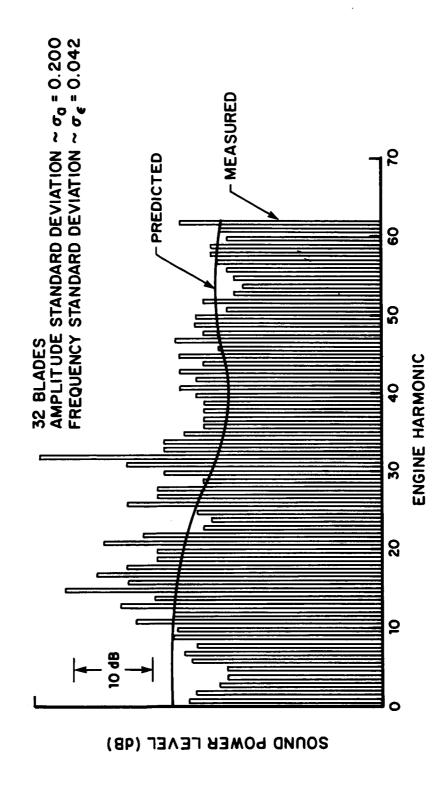


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COMPARISON OF MEASURED AND PREDICTED MPT SPECTRA (BURDSALL et al.). FIG. D-2

APPENDIX E

ALGORITHM FOR DERIVATION OF STATOR LEADING EDGE TRACE VELOCITY IN STATOR FIXED COORDINATES

APPENDIX E. ALGORITHM FOR DERIVATION OF STATOR LEADING EDGE TRACE VELOCITY IN STATOR FIXED COORDINATES

The geometry of a rotor wake as it reaches the stator is given in Fig. E-1. Refer to Fig. 9b in the text for 3-dimensional representation. The following four steps give the rotor wake shape and local trace velocity for both an unswept stator (or at the inlet plane of a swept stator), and for swept stators. Aerodynamic reaction on the rotor path by the stator is not taken into account.

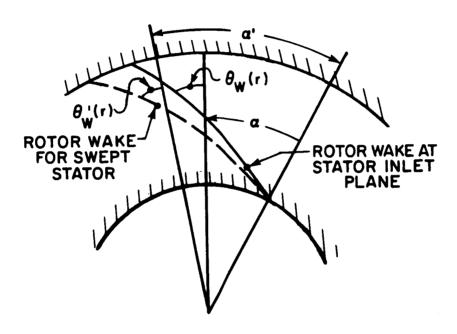


FIG. E-1. GEOMETRY FOR CALCULATION OF ROTOR WAKE SHAPE AND TRACE SPEED ON STATOR VANES.

1) Unswept, Unskewed Stator

The skew of the rotor wake at stator plane is $\alpha(r)$. The local angle between the wake and the radial direction may be derived from:

$$\tan \theta_{w} = r \frac{\delta \alpha}{\delta r}$$

The trace velocity in the radial and axial directions is respectively

$$V_{T_{B_R}}(r) = (wr)/tan_w(r)$$

and

$$V_{T_{B_{\mathbf{X}}}} = 0$$

where:

V_{TBR,x} = the trace velocity in blade fixed coordinates for radial and axial directions respectively.

2) Add Sweep μ_{R}

The rotor wake in the conical surface of the swept leading edge is changed, as follows.

$$V_{TB_R}(r) = \frac{wr}{\tan\theta'_w(r)}$$
 where $\tan\theta'_w = r \frac{\delta\alpha'}{\delta r'}$

and

$$V_{T_B}^{\dagger}(r) = V_{T_B}$$
 $tan\mu_B(r)$ where $\mu_B(r) = local blade sweep angle.$

$$\alpha'(r) = \alpha(r) + \Delta x \frac{\delta \alpha(m)}{\delta x}$$
,

where Δx is the downstream displacement of the leading edge caused by sweep.

$$\tan \theta'_{W}(r) = r \frac{\delta \left[\alpha(r) + \Delta x \frac{\delta \alpha(r)}{\delta x}\right]}{\delta r}$$

$$= r \frac{\delta \alpha(r)}{\delta r} + r \Delta x \frac{\delta^{2} \alpha(r)}{\delta x \delta r} + r \frac{\delta \Delta x}{\delta r} \frac{\delta \alpha(r)}{\delta x}$$

$$= \tan \theta_{W} + \Delta x \frac{\delta \tan \theta_{W}}{\delta x} + r \frac{\delta \alpha(r)}{\delta x} \tan \theta_{B}$$

$$\tan \theta'_{W}(r) = \tan \theta_{W} + r \frac{\delta \alpha(r)}{\delta x} \tan \theta_{B}$$

3) Trace Velocity for Swept Stator in Stator-Fixed Coordinates

The radial component of trace velocity is

$$V_{TB_{R}}(r) = \frac{wr}{\tan \theta_{w}^{*} + r \frac{\delta \alpha(r)}{\delta x} \tan \mu_{B}}$$

where $tan\theta_{W}^{*}(r,x) = tan\theta_{W} + \Delta x \frac{\delta tan_{W}}{\delta x}$.

The axial component is:

$$V_{TB_{x}}(r) = \frac{wr \tan \theta_{B}}{\tan \theta_{w}^{*} + r \frac{\delta \alpha(r)}{\delta x} \tan \theta_{B}}$$

where θ_W^* is determined from cross-plots of the wake path in the r, α plane at various axial locations, and $\delta\alpha/\delta r$ = the local wake helix pitch angle determined from wake crossplots in the meridional plane.

4) Transformation of Trace Velocity Amplitude From Stator-Fixed Coordinates to Gas-Fixed Coordinates

where v_{G} is the gas velocity in the axial and circumferential x,c directions, respectively.

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LIST OF SYMBOLS

A	=	axial
Ain	=	inlet area (to rotor passage)
A _{min}	=	geometric throat area
As	=	area to choke the flow
a_0 t/ λ	=	normalized distance
b	=	blade semichord
В	=	number of blades
c	=	chord length; sound speed
C ₀	=	phase or trace velocity; sound speed
CG	=	center of gravity
$^{\mathrm{c}}{}_{\mathrm{L}}$	=	lift coefficient
D	=	diffusion factor
DCA	=	double circular area (blade profile)
đ	=	distance from blade section c.g. to pressure surface; circumferential spacing between adjacent stator tips
dl	=	swept leading edge element
E	=	acoustic energy (radiated from a single SBLE tip)
f	=	Mach factor (= $1/M_{W_1L}$); frequency
Lie		

= blade passage frequency of rotor blade

 $\mathbf{f}_{\mathbf{r}}$

$\Delta extsf{F}_{ extsf{j}}$	=	centrifugal force at center of a blade volume element located at $\mathbf j$
G ₂ (m)	=	function of flow Mach number
h	=	unit impulse response function
I	=	acoustic intensity
j	=	source number
k	=	wavenumber; constant
k _r	=	radial wavenumber
L	=	harmonic
L	=	distance from section c.g. to leading edge; fluctuating lift
LCF	=	low cycle fatigue
LE	=	leading edge
$^{ m L}_{ m mn}$	=	rotor/stator interaction harmonic
m	=	circumferential mode number; component of Mach number; function in deviation angle formula
M	=	Mach number; moment
M _w	=	relative flow Mach number component
M _{w 1}	= '	relative inlet Mach number
M v W 1	=	Mach number required for a subsonic edge to achieve sonic (= M_{w_1})
M _{w₁L}	=	<pre>component of Mach number which is always normal to the leading edge (= 1 for sonic LE; < 1 for subsonic LE</pre>

M _{W1L}	=	Mach number required to make a subsonic LE a sonic LE
$^{\Delta M}$ ij	=	moment of j force about c.g. of section i
N	=	rotation rate
n	=	shape parameter for polynomial blade forms; harmonic number
P	=	total pressure; static pressur
P or p	=	far field acoustic pressure
P	=	location of leading edge point
P/P	=	pressure ratio
P	=	static pressure after normal shock
PNL	=	Perceived Noise Level
q	=	source strength
A	=	monopole source strength per unit length
R	=	distance from origin
R _c	=	radius of curvature of streamline
r	=	radial distance
s	=	circumferential blade spacing
SAP	=	Structural Analysis Program
SCF	=	Stress Concentration Factor
SBLE	=	Stator Blade Leading Edge
SR	=	sweep reversal

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t thickness; time t(x)thickness distribution Т standard deviation (time) TE trailing edge U or u air velocity; wheel speed velocity v peak velocity defect v_0 V number of stator vanes; velocity ΔV, volume element of a blade W mass flow rate; velocity; inlet relative velocity $\overline{\mathbb{W}}$ average velocity w_{as} specific mass flow w/w flow deceleration rate W relative velocity Δx downstream displacement of SBLE caused by sweep linear distance x z, z_{T} axial coordinate of leading edge

S-4

GREEK

α	=	angle of attack; angle between radial wave vector k and radial flow vector u; local Mach angle; wake displacement angle from radial
α 3	=	stator inlet angle
β	=	relative flow angle; exit angles of flow
Υ	=	setting angle; ratio of specific heats
δ .	=	flow deviation angles; Dirac delta function
δ ₃	=	stator deviation angle
ε	=	slope of the relative flow velocity; slope between lines connecting section LE and CG and a line connecting LE with lower surfaces coordinate at mid-chord ($\varepsilon_{w_1} = \sin^{-1}V_r/w_1$)
$\epsilon_{ exttt{j}}$	=	centerline-projected displacement of the c.g.'s of an airfoil section at r, relative to one at r,
λ	=	acoustic wavelength
λ	=	acoustic wavelength at blade passage frequency
η	=	polytropic state efficiency
θ	=	circumferential angle
$\theta_{\mathbf{w}}$	=	angle between radius and rotor wake centerline, unswept stator
$\theta_{\mathbf{w}}$ '	=	angle between radius and rotor wake centerline, swept stator

$\Theta_{\mathbf{L}}$	=	angular abscissa of leading edge point
μ	=	Mach cone angle; radial order of acoustic modes
$^{\mu}$ B	=	stator sweep angle
μ"	=	projection of the Mach cone angle on the w-r plane
ν	=	hub-to-tip ratio; lateral sweep angle; section thickness ratio (t _{max} /c); relative thickness
π	=	acoustic power
ρ	=	density
σ	=	sweep angle; stress; cascade solidity
τ	=	shear stress; LE and TE thickness factors; time interval between successive events
^τ b	=	time for a gust to travel the swept semi- chord (b)
ф	=	local camber angle; angle between leading and trailing edge along the unwrapped conical surface
Φ	=	mean camber angle
ω	=	radian frequency; wheel rotation speed
$\omega_{f r}$	=	radian frequency of blade passages
Ω	=	shaft rotation frequency

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SUBSCRIPTS

A	=	axial; along blade leading edge
a	=	acoustic
ax	=	axial
В	=	blade-fixed coordinates
С	=	circumferential
crit	=	critical
cg	=	center of gravity
D	=	defect
G	=	gas-fixed coordinates
g	=	geometric
i	=	component parallel to blade array
i, j	=	indices of spatial coordinates
L	=	normal to leading edge; leading edge
L	= .	lower
M	= .	mainstream
m	=	meridional component
minm	=	minimum
0	=	trace speed
R	=	radial

r	=	radial; component normal to blade array
T	=	tangential; wake trace
t	=	tip
u	=	upper; tangential
W	=	tangential
x	=	chordwise distance from LE; direction normal to z-axis
У	=	Cartesian coordinate normal to z-axis
Z	= '	axial
∞	=	freestream relative
1,2,3	=	x, y, and z directions
1	=	rotor inlet station
2	=	rotor exit station
3	=	stator inlet station
4	=	stator exit station