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Transient Catalytic Combustor Model

James S. T'ien Case Western Reserve University

May 1981

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Prepared for National Aeronautics and Space Administration Lewis Research Center Under Grant NSG-3230

for U.S. DEPARTMENT OF ENERGY Conservation and Renewable Energy Office of Vehicle and Engine R&D

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TABLE OF CONTENTS

ABSTRACT	1
INTRODUCTION	2
Chapter I. Transient Catalytic Combustor Model	4
I.1 Nonsteady Combustion Model	4
Consideration of Time Scales Quasisteady Gas Phase Unsteady Solid with Thin-wall Substrate Initial and Boundary Conditions Numerical Solution Procedure	4 5 9 11 12
I.2 Computed Results	13
Steady State Transient Response	13 19
Chapter II. Steady-State Computation and Comparison with Experiments	29
II.1 Three-step Semi-global Gas-phase Chemical Reactions	29
II.2 Computed Results	34
Effects of Catalytic Bed Length, Downstream Reaction Distance, Reference Velocity and Adiabatic	36
Flame Temperature Effect of Inlet Temperature and Pressure	43
SUMMARY	47
REFERENCE	49
NOMENCLATURE	51
Appendix I Listing of Computer Programmes for Chapter I.	55
Appendix II Listing of Computer Programmes for Chapter II.	72

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PAGE

ABSTRACT

A lean combustion model for monolithic catalytic combustors is given. The model, consisting of several semi-global chemical reaction steps in the gas phase and on the surface, is capable of analyzing CO and THC emissions and combustor efficiency in both steady and unsteady states.

In the steady-state model computation presented, the influence of operating and design parameters on the minimum combustor length is studied. Special attention is given to the effect of after-bed gas phase reaction space. Comparison with experimental data by Anderson indicates good agreement in the range of parameters covered.

In the transient analysis, a quasisteady gas phase and a thermallythin substrate are assumed. The combustor response delay is due to the substrate thermal inertia. Fast response is found to be favored by thin substrate, short catalytic bed length, high combustor inlet and final temperatures and, in most cases, small gas channel diameters. The calculated gas and substrate temperature time history at different axial positions provide an understanding of how the catalytic combustor responds to an upstream condition change. The computed results also suggest that the gas residence times in the catalytic bed and in the after-bed space are correlatable with the nondimensional combustor response time. When fast transient responses are required, both steady and unsteady studies have to be made to achieve a meaningful compromising design.

INTRODUCTION

The operation and the design of catalytic combustors are limited on one end by the high temperature that the catalysts can tolerate and on the other end (low temperature) by the acceptable emission levels of CO and unburnt hydrocarbons. The aim of the combustion engineer is to design a compact and durable combustor capable of efficient and clean combustion. Catalytic combustor modeling can contribute to this objective by providing more understanding to the detail processes occurring in the reactor and to serve as a design guidance for minimizing the numbers of experimental tests needed.

There are several steady-state catalytic combustor models in existence (Ablow and Wise, 1979, Cerkanowicz, et al, 1977,Kelley et al, 1977, Kendall et al, 1979) and they show a high degree of success in describing the events occurring in the combustor. In particular, the model by Kelley et at (1977) containing detailed gas phase chemistry for methane/air reactions can predict pollutant emissions. Detailed kinetics, unfortunately, are not yet available for most other practical fuels. Since emission characteristics are important considerations for catalytic combustor design, in the absence of more detailed information, models consisting of several key semi-global chemical steps seem appropriate.

In addition to the emission characteristics, the transient behavior of catalytic combustors is expected to be different from that of a conventional gas turbine burner, due to the large thermal inertia of the substrate. Non-steady operations are obviously important to transportation engines since their power levels have to be changed frequently. Transients may also be a concern with stationary gas turbine applications, since the

ignition/shutdown operation can produce excessive thermal stress or thermal shock if the combustor is not designed or operated properly and the substrate can fail as a result (DeCorso and Carl, 1979).

Typical questions concerning transient operations include the combustor response time, the type of response, the unsteady substrate temperature history and the resultant thermal stress distribution. At the present time, both experiment and theory on transient catalytic combustion are lacking. A model is presented here attempting to analyze the problem theoretically. In Chapter I of this report, the transient combustion model will be formulated and numerical solution will be presented as a parametric study. Since the transient computer code can also yield steady-state results, in Chapter II, a detailed computation of steady states is given and a comparison with Anderson's data is made. As far as the combustion models are concerned, the one in Chapter II has a three-step global gas-phase chemical reaction vs. a two-step one in Chapter II. The reason for using the three-step scheme is discussed in the beginning of Chapter II.

CHAPTER I TRANSIENT CATALYTIC COMBUSTOR MODEL

I.1 Nonsteady Combustion Model

Consideration of Time Scales

For a monolithic catalytic combustor with uniform cell distribution in the plane normal to the approaching flow direction, if the combustor wall is well insulated, then the study of the whole reactor can be reduced to the consideration of a single cell unit. Referring to Fig. I.l, each cell unit consists of an open gas channel, the associated substrate volume and an after-bed gas reaction space.



Figure I.1 Schematic Drawing of One Cell Unit

Considering first the steady-state combustion, the relevant time scales are the gas residence time in the reactor, the heat and mass transfer time between the gas and the solid surface inside the channel and the gasphase and surface reaction times. When a transient is caused by an upstream parameter variation, then the new time scales involved include the input characteristic time, the time for the temperature wave to reach the central plane of the substrate and the time for heating-up (or cooling-down) the substrate.

Table Ll defines these characteristic times and their estimated magnitudes. It should be noted that some of the time scales vary along the channel of the reactor bed. For example, the heat/mass transfer time and the substrate heat-up time are shorter at the flow entrance region than those for far downstream, and the ones used in Table I are those based on fully-developed flow profiles. However, the estimation from Table I does serve the purpose of order-of-magnitude comparison. From this table we see that the longest time is the solid heat-up time, being of the order of seconds. If the time scale we are interested in is much greater than the gas residence time (~15 msec), then the gas phase processes (including heat/mass transfer and chemical reactions) can be regarded as in a quasisteady state. If furthermore the substrate half-thickness is smaller than about .2 mm, then the substrate temperature distribution in the direction normal to the channel axis can be regarded as uniform at any given time. In this limiting case then, a "quasi-steady-gas-phase and thermallythin-substrate" model is applicable with the only transient process being the substrate heating-up or cooling-down. In the following section, this model will be described in mathematical form.

Quasisteady Gas Phase

Quasisteady gas phase implies that the differential equations for the

Table I.J	. Estimate	of	Transient	Time	Scales	in	Catal	ytic	Monolith	Combustor
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Time	Definition	Estimated Magnitude
Gas residence time in reactor bed	$(^{L*/u_{ref}^{*}}) (^{A*/A_{T}^{*}})$	1 - 15 msec
Gas residence time in after-bed space	L* /u* as ref	0 - 15 msec
Heat/mass transfer time between gas and solid surface in channel	$\frac{1}{\mathrm{Nu}_{\infty}} \frac{1}{\pi} \frac{d^{\star^2}}{4\alpha^{\star}(0,0)}$.5 - 7.5 msec for d* ≤ 3.6 mm
Chemical reaction times (gas-phase and heteroge- neous)		Same order of magnitude as gas residence times
Time for temperature wave to reach the substrate center plane	$\frac{d^{*2}_{s}}{\frac{\sigma^{*}_{s}}{s}}$	0.5 - 25 msec for $d^* \leq .2 \text{ mm}$
Substrate heat-up time	$\frac{1}{\mathrm{Nu}_{\infty}}\frac{1}{\pi}\frac{\overset{A*\rho*C*}{s}\overset{c}{s}\overset{c}{s}}{k^{*}(0,0)}$	0.5 sec - 20 sec
Transient input time		variable, specified

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gas-phase processes are the same as those for the steady state. Only the boundary conditions on the solid surface are different. There are several steady-state catalytic combustor models in existence [Cerkanowicz et al, 1977, Kelly et al, 1977, 1979] and they show a high degree of success in describing the events occurring in the combustor. In particular, the model by Kelley et al (1977) containing detailed chemistry for methane/air reactions can predict pollutant emissions. Detailed kinetics, unfortunately, are not yet available for most other practical fuels. Since emission characteristics are important considerations for catalytic combustor design, in the absence of more detailed information, a model consisting of several key semi-global chemical steps is proposed.

For fuel-lean catalytic combustion using nitrogen-free hydrocarbon fuels, the NO_x emission is negligible because of low flame temperatures. The only pollutants needed to be considered are unburnt hydrocarbons (UHC) and carbon monoxide (CO). For this reason, the two semi-global chemical reactions used in the gas phase are:

(A)
$$C_{n m}^{H} + 1/2 (n + m/2) O_{2} \rightarrow n CO + m/2 H_{2}^{O}$$

$$\frac{d[C_{n m}]}{d t^{*}} = -C_{1}^{*} p^{*\alpha_{1}} [C_{n m}^{H}]^{\alpha_{2}} [O_{2}]^{\alpha_{3}} T^{*\alpha_{4}} \exp(-E_{1}^{*}/RT^{*})$$
(1)
(B) $CO + 1/2 O_{2} \rightarrow CO_{2}^{O}$

 $\frac{d[CO]}{dt^{*}} = -C_{2}^{*} p^{*\beta_{1}} [CO]^{\beta_{2}} [O_{2}]^{\beta_{3}} [H_{2}O]^{\beta_{5}} T^{*\beta_{4}} exp (-E_{2}^{*}/RT^{*})$ (2)

In describing the flow in the reactor channel, we follow previous investigators in using the plug flow approximation. For transient cases, in general, the fluid properties are functions of x, the distance along the channel axis and t, the time. For example, density of the gas will be

expressed as $\rho(x,t)$. The nondimensional forms of these equations are summarized in the following (supscript * denotes dimensional quantity and without * denotes nondimensional quantities).

Continuity:
$$\rho u = \rho(o,t) u(o,t)$$
 (3)

Momentum:
$$p = p(o,t)$$
 (4)

Energy:
$$\rho u \frac{\partial T}{\partial x} + J_{H}(T - T_{s}) = q_{1}Y_{HC}(0,0) w_{1} + q_{2}Y_{HC}(0,0) w_{2}$$
 (5)

Species:

Hydrocarbon:

$$\rho u \frac{\partial \mathcal{Y}_{HC}}{\partial x} + J_{D_1} (\mathcal{Y}_{HC} - \mathcal{Y}_{HC}, s) = -w_1 \qquad (6)$$

Carbon monoxide:

$$pu \frac{\partial \mathcal{Y}_{CO}}{\partial x} + J_{D_2}(\mathcal{Y}_{CO} - \mathcal{Y}_{CO,S}) = w_1 c_n - w_2$$
(7)

Oxygen:

$$\mathcal{Y}_{O_2} = 1 - \phi(1 - \mathcal{Y}_{HC}) + \frac{1}{2} \qquad \mathcal{Y}_{CO} \quad \left(\begin{array}{c} \frac{Y_{HC}(0,0)}{Y_{O_2}(0,0)} & \frac{W_{O_2}}{W_{CO}} \right)$$
(8)

v

(9) Equation of state: $p = \rho T$ The reaction rates are given by

$$w_{1} = B_{1} p^{\alpha_{1}} \rho^{(\alpha_{2} + \alpha_{3})} T^{\alpha_{4}} \mathcal{Y}_{HC} \mathcal{Y}_{02} e^{-(E_{1}/T)}$$
(10)

$$w_{2} = B_{2} p^{\beta_{1}} \rho^{(\beta_{2}+\beta_{3}+\beta_{5})} T^{\beta_{4}} \mathcal{Y}^{\beta_{2}}_{CO} \mathcal{Y}^{\beta_{3}}_{O_{2}} Y^{\beta_{5}}_{H_{2}O} e^{-(E_{2}/T)}$$
(11)

and
$$Y_{H_20}(x,t) = \frac{m}{2} \frac{W_{H_20}}{W_{HC}} Y_{HC}^{(0,0)} \begin{bmatrix} \mathcal{Y}_{HC}(o,t) - \mathcal{Y}_{HC}(x,t) \end{bmatrix}$$
(12)

The definition of B_1 and B_2 can be found in the nomenclature.

In deriving Eqns (5-8), heat conduction and mass diffusion in the axial direction are neglected because the Peclet number based on typical gas velocity (>10 M/sec) is much greater than unity [Ablow and Wise, 1979]. The dimensionless lateral heat and mass transfer coefficients J_H and J_{Di} , in Eqns (5-7) are derived from Nusselt number calculation in entrance flow in a tube of constant surface temperature [Kays, 1966]. Following Kelly et al (1977), a modification is made at the entrance point based on local stagnation flow estimation. The use of entrance flow transport properties, rather than fully-developed constant values, make a great difference in the temperature and species distributions in the flow entrance region.

The above system of equations can also be applied to the after-bed space where only gas-phase reactions occur. This is done by putting J_{H} and J_{Di} equal to zero in Eqns (5-7) and by properly changing the mass flux per unit area in Eq (3) due to area change.

Unsteady Solid with Thin-Wall Substrate

When the substrate wall is thermally thin, the solid temperature can be regarded as a function of axial distance and time only, i.e., $T_s(x,t)$. The energy balance of the solid, including the solid surface, results in the following equation:

$$A_{s}^{*} \rho_{s}^{*} C_{s}^{*} \frac{\partial T_{s}^{*}}{\partial t^{*}} - h_{T}^{*} S^{*} (T^{*} - T_{s}^{*}) = S^{*} [q_{3}^{*} J_{3}^{*} + q_{4}^{*} J_{4}^{*}]$$
(13)

In Eq (13) the heat conduction in the axial direction in the solid is neglected in comparison with the heat transfer rate between the gas and solid across the gas channel.

Two semi-global catalytic surface reactions are assumed. They are

(C)
$$C_{nm}^{H} + (n + \frac{m}{4}) O_2 + nCO_2 + \frac{m}{2} H_2^{O}$$
 (14)

with the corresponding surface reaction rate J_3^{\bigstar} given by

 $J_{3}^{*} = C_{3}^{*} [C_{n}H_{m}]_{s} e^{-E_{3}^{*}/RT_{s}^{*}}$ and (D) CO + $\frac{1}{2} O_{2} \rightarrow CO_{2}$ (15)

with
$$J_{4}^{*} = C_{4}^{*} [CO]_{s} e^{-E_{4}^{*}/RT_{s}^{*}}$$

In surface reaction (C), the hydrocarbon is oxidized to form CO_2 and H_2O , not CO such as in gas-phase reaction (A). This is based on the present available experimental evidence in oxygen-rich systems, only CO_2 , not CO, is found on or close to the catalytic surface [Schwartz et al, 1971, Anderson, 1976 and Marteney, 1979].

Eq (13) is non dimensionalized by first defining a proper reference time scale τ^* .

$$\tau^{*} \equiv \frac{1}{Nu_{\infty}} \frac{1}{\pi} \frac{A_{s}^{*} \rho_{s}^{*} C_{s}^{*}}{k^{*}(0,0)}$$
(16)

and

$$\mathbf{r}_{nk} \equiv \frac{\mathbf{N}\mathbf{u}_{x}}{\mathbf{N}\mathbf{u}_{\infty}} \frac{\mathbf{k}^{*}}{\mathbf{k}^{*}(0,0)}$$
(17)

Eq (13) becomes

 $t \equiv t^*/\tau^*$

$$\frac{\partial T_{s}}{\partial t} - r_{nk} (T - T_{s}) = B_{3}Y_{HC}(0,0) q_{3}^{\rho} \mathcal{Y}_{HC,s} \exp(-E_{3}/T_{s}) + B_{4}Y_{HC}(0,0)q_{4}^{\rho} \mathcal{Y}_{CO,s} \exp(-E_{4}/T_{s})$$
(18)

Neglecting transient accumulation on the surface, the surface hydrocarbon and carbon monoxide are given by

$$r_{nk} \left(\mathcal{Y}_{HC} - \mathcal{Y}_{HC,s} \right) = \frac{1}{(L_{e_1})^{2/3}} B_3 \rho \mathcal{Y}_{HC,s} \exp(-E_3/T_s)$$
 (19)

$$r_{nk}(\mathcal{Y}_{CO} - \mathcal{Y}_{CO,s}) = \frac{1}{(L_{e_2})^{2/3}} \mathcal{B}_4 \rho \mathcal{Y}_{CO,s} \exp(-E_4/T_s)$$
(20)

Initial and Boundary Conditions

The system of equations (3-12, 18-20) needs initial and boundary conditions. Initial conditions (at t = 0) are required for T_s , (\mathcal{G}_{HC})_s (\mathcal{G}_{CO})_s as a function of x and upstream boundary conditions (at x = 0) should be specified for \mathcal{G}_{HC} , \mathcal{G}_{CO} , \mathcal{G}_{O2} , T, p and u as a function of time. These conditions depend on the types of transient input which can vary with engine designs and mode of operations. In the present paper, a simple start-up transient is investigated, i.e. catalytic combustor response to a step-wise fuel injection.

The substrate temperature at time t=0 is specified. This temperature can be the result of external heating of the substrate, say, by a torch. The fuel and air are then supplied at upstream. The model is intended to describe the events from t=0 to the final steady state. Mathematically, the conditions used are

 $T_{s}(x,0) = \text{specified}$ $\mathcal{Y}_{HC,s}(x,0) = 0$ $\mathcal{Y}_{CO,s}(x,0) = 0$ T(0,t) = 1 $\mathcal{Y}_{HC}(0,t) = 1$ $\mathcal{Y}_{CO}(0,t) = 0$ $\mathcal{Y}_{02}(0,t) = 1$ u(0,t) = 1

Numerical Solution Procedure

Although most dependent variables in this problem are function of both x and t which, in general, results in partial differential equations, the assumption of quasisteady gas phase and thermally thin substrate greatly simplifies the mathematical property of the system and results in a simpler solution procedure.

(21)

The gas-phase differential equations (5-7) have only first derivative in x with t as a parameter, the solid equation (18) has only derivative in t with x as a parameter; therefore, they can be integrated at each time (t) or position (x) as ordinary differential equations using Runge-Kutta method. The procedure is started by first integrating Eqns (5-7) forward in x from upstream to the end of combustor for T, \mathcal{Y}_{HC} , \mathcal{Y}_{CO} using the solid surface

quantities specified by the initial conditions, then Eq (18) is integrated for one time step (Δ t) to find T_s(x, Δ t) and Eqns (19,20) are solved algebraically to find $\mathcal{Y}_{HC,s}(x, \Delta t)$ and $\mathcal{Y}_{CO,s}(x, \Delta t)$. These surface quantities are substituted back to Eqns (5-7) to start another cycle of integration. Steady-state and transient solutions are obtained using the same code. The computational time ranges from 10-50 seconds in a VAX-11 computer. The numerical scheme is an efficient one and it is suitable for parametric studies.

I.2 Computed Results

Steady State

Selected steady-state results will be presented first to facilitate the discussion on the transient response. Specifically, the relative importance of the catalytic bed length and the after-bed-gas-phase-reaction distance will be shown. In previous theoretical models, emphasis was placed on the combustion inside the catalytic bed only. The contribution of gasphase reactions in the downstream space after the bed has been demonstrated in the experiments by T'ien and Anderson (1979) and Anderson (1980). Utilization of the pure gas-phase reactions in the downstream of the monolith to replace a portion of the catalytic bed will be shown to be crucial in reducing the catalytic combustor transient response time.

Table I.2 lists the values of chemical kinetic constants for reactions A to D used in this chapter. The hydrocarbon fuel chosen is propane. Semiglobal kinetics for hydrocarbon and CO oxidation are from Edelman (1969) and Dryer and Glassman (1973) respectively, with the pre-exponential factors

Reaction (A)	Reaction (B)	Reaction (C)	Reaction (D)
$C_1^{\star} = 1.5 \times 10^5$	$C_2^* = 0.71 \times 10^{14}$	$C_3^{\star} = 2.5 \times 10^3$	$C_4^* = 10^5$
$\alpha_{1} = 0.3$	$\beta_1 = 0$	$E_3^* = 10$ (kcal/mole)	E* = 17.8 (kcal/mole)
$\alpha_{2} = 0.5$	$\beta_2 = 1$		
$\alpha_3 = 1$	$\beta_3 = 0.25$		
α ₄ = 1	β ₄ = 0		
E [*] = 24 (kcal/mole)	$\beta_{5} = 0.5$		
	$E_2^{\bigstar} = 40$ (kcal/mole)		

Table I.2 Chemical Kinetic Constants for Chapter I

adjusted. The activation energies for reactions C and D are obtained from the data of Marteney (1979) and Kuo and Morgan (1971) respectively with adjusted pre-exponential factors. It should be noted that the surface reaction rates contain information on catalyst type and loading density as well as the washcoat properties.

The design parameters considered are d*, channel hydraulic diameter, A^*/A_T^* , the percentage of open area (or equivalently, A^*/A_S^* , the ratio of open-to-close areas), L*, the catalytic bed length and L_{as}^* , after-bed gas-phase reaction length. The minimum catalytic combustor length required to reach the emission goal in steady-state operations is denoted using the subscript EG. The emission goal is chosen to be 1.64 g/kg of fuel for UHC and 13.6 g/kg of fuel for CO [Anderson, 1977]. The operating parameters include p*, the pressure, T_{in}^* , combustor inlet temperature, T_{af}^* , adiabatic flame temperature (or equivalently the fuel/air equivalence ratio ϕ) and U_{ref}^* , the reference velocity of the approach gas upstream of the reactor bed.

Fig. I.2 gives the profiles for one sample calculation. In this case, the propane fuel is oxidized quickly on the catalyst surface upon entering into the reactor channel and resulting in a high surface temperature, the gas temperature is raised by heat transfer from the substrate, gas-phase oxidation of propane is accelerated to form CO, and CO, in turn, is oxidized by both gas-phase and surface reactions to CO_2 . Actually, two cases are shown in Fig.I.2. In one case, the reactor bed length is 4 cm, in the other, 8 cm. When the bed length is cut short at 4 cm, the surface oxidation reactions for the hydrocarbon fuel and CO stop there. This results in a higher CO concentration as shown by the dotted curve. The hydrocarbon curve





also shows a difference after 4 cm but the difference is too small to be shown clearly in this figure. The gas temperature is also lower in a portion of the combustor for the 4 cm case.

Fig. I.3 gives the combustor residence time required to reach the emission goal vs. the residence time in the catalytic bed. The "catalytic bed residence time", t_b^* , is defined by $L^*A^*/U_{ref}^* A_T^*$, where U_{ref}^* is the gas velocity upstream of the bed and the gas velocity inside the catalytic channel at the channel entrance is $U_{ref}^*A_T^*/A^*$, due to the area change. To be exact, the gas residence time in the bed should be $\int_0^{L^*} 1/U^* dx^*$ since the gas velocity in the channel, U^* , is changing continuously downstream due to the heat release. However, since $U^*(x)$ is not known until the solution is achieved, the catalytic bed residence time, t_b^* , as defined in Fig.I.3 is a good reference quantity and it is with this understanding when we use the terms of "bed residence time" and "combustor residence time".

Referring to the curve and the calculated points corresponding to $T_{in}^{\star} = 1000 \text{ K}$ in Fig.I.3 we see that the residence times provide an excellent correlation for different values of U_{ref}^{\star} , L^{\star} , A^{\star}/A_{T}^{\star} and the required L_{as}^{\star} to reach emission goals. As the catalytic bed residence time, t_{b}^{\star} , increases, the total combustor residence time required to reach emission goals, t_{EG}^{\star} , first decreases sharply and then levels off. The two other curves in Fig. I.3 give the effects of different combustor inlet temperatures. Higher inlet temperature required lower combustor residence time to reach emission goals for a fixed catalytic bed residence time. At 1200 K inlet temperature, no catalytic element is needed; emission goals can be reached in an acceptable gas residence time for a plug flow combustor without flame stabilizer (t_{b}^{\star} = o and T_{EG}^{\star} = 14 seconds). This has been verified by the recent



Fig. I.3 Minimum total combustor gas residence time for reaching emission goals (t*) vs. reactor bed residence time (t*) as a function of combustor inlet temperature in steady-state operation. $p^* = 3$ atm, $d^* = 1.8$ mm and $T^*_{af} = 1370$ K.

experimental results of Anderson (1980b). Anderson's study also suggested the correlation using gas residence times (1980a).

Transient Response

One of the major concerns of the catalytic combustor application to transportation gas turbine engines is that the response time might be long. The computed transient results to be presented are combustor response to a stepwise increase of fuel flow rate (zero to a fixed value). Its purpose is to guide the search for "fast-response" catalytic combustor designs.

Some indication of the order of magnitude of the response time can be obtained from the definition of the characteristic time τ^* as defined in Eq (16). Small τ^* requires small thermal inertia, A_{SSSS}^* . If the substrate density and heat capacity are fixed, small τ^* can be achieved with small solid crossectional area A_S^* , see Fig·I·1 for definition of A_S^* . It should be noted that there are other parameters such as catalytic bed length and channel diameter which will influence the response time and they are not contained in τ^* . This will be discussed later as we see the computed results.

The combustion efficiency at the end of the combustor will be used to characterize the combustor response time. There are two ways to define the combustion efficiency. One is the so-called "carbon-balanced" efficiency, η_{CB} [Anderson, 1975], where the combustion inefficiency is measured by the emission levels of CO and UHC and the energy carried away by them. The other efficiency is the ordinary one defined by the enthalpy rise across the combustor divided by the chemical energy available. If the gas specific heat is approximated as a constant, this efficiency, η_{T} , is given by $(T^* - T^*_{in})/(T^*_{af} - T^*_{in})$ where T* is measured at the same location as the efficiency. For a perfectly insulated catalytic combustor in steady state,

 n_{CB} and n_{T} are the same. However, the two are different in transient operations when there is a thermal lag in the substrate.

For the following transient calculation, the initial surface temperature condition is given by $T_{c}(x,0) = 1$, see Eq (21). For 1000°K inlet temperature, two bed lengths, 4 cm and 8 cm, are chosen for the transient calculation and efficiency is evaluated at their individual minimum combustor lengths according to Fig.I.3. Fig.I.4 shows the combustion efficiencies as a function of non-dimensional time. Looking at the curves for $\boldsymbol{\eta}_{\mathrm{T}}$ (efficiency based on temperatures), the shorter bed (4 cm) combustor has a faster response than that of the longer bed (8 cm). If we take the time for $\eta_{\rm T}$ to reach 80% as an indication of characteristic response time, for the 4 cm bed, $t_{ch} = 1.5$ and for the 8 cm bed, $t_{ch} = 2.5$; namely, the response time of 4 cm bed combustor is only 60% of that of the 8 cm bed combustor. From the consideration of steady-state operation alone, one would choose the 8 cm catalytic bed since, as can be deduced from Fig.I.3, this produces the shortest required combustor length (11 cm). However, combining transient and steady-state considerations, the 4 cm bed combustor may be a better choice as its response time is 60% of that of the 8 cm bed, with the total combustor length only slightly longer.

To understand why longer reactor bed results in a longer response time, surface and gas temperature distributions are plotted in Fig. I.5 as a function of time. From the substrate temperature distribution we see that during start-up transient, the solid temperature is higher in the flow entrance region as a result of higher mass transport and surface reaction rates. Comparing the solid and gas temperature histories for the case when catalytic bed length is 8 cm, we see that at t = 0.5, the solid is



Fig. I.4 Combustion efficiencies as a function of time during start-up transient. $p^* = 3$ atm, $\phi = 0.15$ and $A^*/A_T^* = .667$.



Fig.I.5 Gas and substrate temperature history during transient start-up for two bed lengths. Same operating and design parameters as those in Fig.I.4.

transfering heat to the gas in the upstream region in the bed. At $x^* = 3.5$ cm, the gas temperature becomes higher than the substrate temperature, heat is then transferred from the gas to the solid in the down-stream portion of the reactor channel. If the heat loss rate is greater than heat generation rate due to gas phase reactions, then the gas temperature will drop as shown in Fig.I.5. If we cut the reactor bed short at 4 cm, a portion of this heat loss can be eliminated and the downstream gas temperature ($x^* > 4$ cm) will be higher as indicated by the dotted curves in Fig.I.5. This explains why a shorter reactor bed can have a faster response.

More computation of the response time dependence on the catalytic bed length, gas velocity and combustor inlet temperature is shown in Fig. I.6. As will be shown later, the nondimensional transient response also correlates well with the gas residence times in the reactor bed and the after-bed space. Therefore the results will be presented using residence times rather than length and velocity. In Fig.1.6, the adiabatic flame temperature, pressure, channel diameter, open area fraction and L^*/U^* are held constant. The solid line in Fig.I.6 gives the response time if the reactor-bed-gas-residence time is the minimum to reach emission goals in steady-state operations. When the combustor inlet temperature is increased, the minimum required reactorbed-residence time is decreased and the combustor response time is shortened drastically. If a more conservative design is used, i.e. longer bed residence time than is required, the combustor response time is increased for a fixed inlet temperature. The increase is very slight in the beginning when the bed residence time is close to the optimum value but becomes larger as the bed residence time increase further. This implies that a conservative



Fig.I.6 Nondimensional response time (n_T goes to 80%) as a function of combustor inlet temperature and reactor bed residence time. $L^*/U^*_{ref} = 15 \text{ msec}, T^*_{af} = 1370 \text{ K}, A^*/A^*_T = .667, d^* = 1.8 \text{ mm} \text{ and}$ $p^* = 3 \text{ atm}.$

design with reactor bed length somewhat longer that that of the optimal one will not greatly increase the combustor response time.

Fig.I.7 gives the nondimensional response time as a function of channel hydraulic diameter and flame temperature. For all the calculations in Fig.I.7 the residence time in the catalytic bed, t_b^* , is held constant at 1 milisecond and the residence time in the after-bed space, t_{as}^* , is 13.5 miliseconds. The computation for 1370 K and 1590 K includes different combinations of L*, L_{as}^* , U_{ref}^* and A*/A* and it can be seen from Fig.I.7 that the nondimensional response time correlates well with the residence times. It can, therefore, be concluded that the primary influence of the catalytic bed and after-bed lengths, the approaching gas velocity and the open area percentage is contained in the gas residence times in the catalytic bed and in the after-bed space for both steady and transient operations.

Fig. I.7 also shows that for a given flame temperature, the nondimensional response time increases with decreasing channel diameter d*. This, however, does not necessarily mean that the dimensional response time increases with decreasing d*. If the open area percentage (or equivalently A^*/A_s^*) is fixed, decreasing d* decreases A_s^* which is used in the reference time scale as defined in Eq (16) and the dimensional response time may actually decrease. Table I.3 gives selected dimensional response times for ceramic and Kanthal metal alloy substrates using the computed results from Fig.I.7. It can be seen that in general the dimensional response time decreases with decreasing channel diameter but the amount of decrease vanishes when the channel diameter becomes too small. In special cases such as the 1590 K flame temperature case, the response time actually reaches a minimum and increases with further diameter decrease.



Fig. I.7 Computed nondimensional catalytic combustor response time correlation with gas residence times for various channel diameters and flame temperatures. $p^* = 3$ atm, $t_b^* = 1$ msec, $t_{as}^* = 13.5$ msec, $T_{in}^* = 1000$ K.

Table I.3 Computed Catalytic Combustor Response Time Using Ceramic and Kanthal Metal Substrates.

 $t_{b}^{*} = 1 \text{ msec}, t_{as}^{*} = 13.5 \text{ msec}, p^{*} = 3 \text{ atm}, T_{in}^{*} = 1000 \text{ K}$

T* (K) af	A*/A* s	d* (mm)	t* (sec) r ceramic	t* (sec) r Kanthal alloy
1370	10	1.8	.848	1.52
1370	10	1.0	.451	.807
1468	10	2.5	.909	1.63
1468	10	1.8	.533	.956
1468	10	1.0	.454	.813
1590	10	2.5	.527	.944
1590	10	1.8	.377	.675
1590	10	1.0	.448	.802
1590	5	1.8	.754	1.35
1590	2	1.8	1.88	3.37

If the gas residence times and channel diameter are held constant, varying the open area percentage (or A*/A*) changes the dimensional response time by a factor of A*/A*. This is the direct consequence of the correlation between nondimensional response time and residence times as just discussed. Large open area percentage (or thin substrate), therefore, favors fast response.

FigI.7 indicates higher flame temperature shortens the response time for large diameter channel but flame temperature makes little difference in response time for very small diameter channels as all curves converge.
CHAPTER II STEADY STATE COMPUTATION AND COMPARISON WITH EXPERIMENTS

II.1 Three-step Semi-global gas-phase Chemical Reactions

Gas phase reaction studies between hydrocarbons and air in lean mixtures have shown (Dryer and Glassman, 1978) that the hydrocarbon fuel first decomposes into a number of lower hydrocarbons with the simulataneous formation of some water vapor. The combination of these "pyrolysis" steps results in an almost isothermal region after which CO is formed and temperature begins to rise. The carbon monoxide is then oxidized and the reactions are completed. These kinetic studies are very useful for the development of semi-global kinetic schemes.

Two recent catalytic combustion models, by Bracco et al (1980) and T'ien (1980), have utilized the gas phase semi-global modeling concept. The study by T'ien was actually for transient response but a quasisteady gas phase was assumed. The gas phase reactions consist of two semi-global steps; the hydrocarbon reacts with oxygen to form CO and H_2O and CO then reacts with oxygen to form CO_2 . On the surface, both hydrocarbon and CO react with oxygen to form CO_2 and or H_2O . The steady-state prediction of that model agrees qualitatively with the experimental trend of CO and hydrocarbon emissions, but a quantitative comparison has not been made. From a pure gas phase reaction point of view, two-step semi-global scheme predicts too high a level of CO in the initial pyrolysis region even though the CO peak level and the decay curve can be made to agree with the experimental data by kinetic parameter adjustment. This early CO appearance problem can be alleviated to a certain

extent by increasing the number of appropriate semi-global gas reaction steps. In the work by Bracco et al (1980), for example, a three-step reaction scheme was used for propane oxidation. Propane (C_3H_8) was assumed to form C_2H_4 first, C_2H_4 to CO next and then CO to CO_2 .

For most practical fuels, the pyrolysis intermediates are numerous, and a dominant intermediate species may or may not exist. Since in high inlet temperature operations ($T_{in}^* > 800^{\circ}K$), the catalytic combustor length required is limited by the emission level of CO, not hydrocarbons (Anderson, 1980a), it seems reasonable to develop a scheme which gives the correct CO characteristics without being tied down by the uncertain nature of the hydrocarbon intermediates.

In this chapter, the steady-state computation will be based on a threestep gas-phase kinetic scheme. The semi-global reactions are:

(A')
$$C_n H_m \rightarrow C_n H_m^{(p)}$$

$$\frac{d[C_n H_m]}{d t^*} = - C_0^* p^{*\gamma_1} [C_n H_m]^{\gamma_2} T^{*\gamma_3} exp(-E_0^*/RT^*)$$
(22)

(A")

$$C_{n}H_{m}(p) + \frac{1}{2} (n + \frac{m}{2}) 0_{2} \rightarrow n C0 + \frac{m}{2} H_{2}0$$

$$\frac{d[C_{n}H_{m}^{(p)}]}{d t^{*}} = - C_{1}^{*}p^{*\alpha_{1}}[C_{n}H_{m}^{(p)}]^{\alpha_{2}}[0_{2}]^{\alpha_{3}}T^{*\alpha_{4}}exp(-E_{1}^{*}/RT^{*})$$
(23)

(B)
$$CO + \frac{1}{2}O_2 \rightarrow CO_2$$

$$\frac{d[CO]}{d t^*} = -C_2^* p^{*\beta_1} [CO]^{\beta_2} [O_2]^{\beta_3} [H_2O]^{\beta_5} T^{*\beta_4} \times exp (-E_2^*/RT^*)$$

(24)

In reaction (A'), Eq. (22), the original hydrocarbon fuel forms the "pyrolyzed hydrocarbon", $C_n H_m^{(p)}$, in an <u>isothermal</u> reaction. The "pyrolyzed hydrocarbon" then reacts with oxygen to form CO and H_2^{O} in reaction (A"). In reaction (B), CO is oxidized to CO_2 . The introduction of reaction (A') serves to decrease the amount of early CO appearance as found using the two-step model. The identity of $C_n H_m^{(P)}$ is of no great consequence for the purpose of this modeling work, if the mass diffusion coefficient is properly evaluated. For convenience, in the present calculation we take $C_n H_m^{(P)}$ to have the same n and m as the original fuel.

Corresponding to the reactions given by Eqs. (22-24), the species equations are:

Original Hydrocarbon:

$$\rho u \frac{d \mathcal{Y}_{HC}}{dx} + J_{D_1} (\mathcal{Y}_{HC} - \mathcal{Y}_{HC,s}) = - w_0$$
(25)

"pyrolyzed" Hydrocarbon:

$$Pu \frac{d \mathcal{Y}_{HC}^{(p)}}{dx} + J_{D1} (\mathcal{Y}_{HC}^{(p)} - \mathcal{Y}_{HC,s}^{(p)}) = w_0 - w_1$$
(26)

Carbon monoxide:

$$\rho u \frac{d \mathcal{Y}_{C0}}{dx} + J_{D2} (\mathcal{Y}_{C0} - \mathcal{Y}_{C0,s}) = w_1 C_n - w_2$$
(27)

Oxygen:

$$\mathcal{Y}_{0_2} = 1 - \phi \left(1 - \mathcal{Y}_{THC}\right) + \frac{1}{2} \quad \mathcal{Y}_{CO} \left(\frac{Y_{HC}(o)}{Y_{0_2}(o)} \frac{W_{0_2}}{W_{CO}}\right)$$
(28)

where

$$\mathcal{Y}_{\text{THC}} = \mathcal{Y}_{\text{HC}} + \mathcal{Y}_{\text{HC}}^{(p)}$$

The reaction rates are given by

$$w_{o} = B_{o} p^{\gamma_{1}} \rho^{\gamma_{2}} T^{\gamma_{3}} \mathcal{G}_{HC}^{\gamma_{2}} \exp(-E_{o}/T)$$
⁽²⁹⁾

$$w_1 = B_1 p^{\alpha_1} \rho^{(\alpha_2 + \alpha_3)} T^{\alpha_4} \mathcal{Y}_{HC}^{\alpha_2} \mathcal{Y}_{0}^{\alpha_3} \exp(-E_1/T)$$
(30)

$$w_{2} = B_{2} p^{\beta_{1}} (\beta_{2} + \beta_{3} + \beta_{5}) T^{\beta_{4}} \mathcal{Y}^{\beta_{2}}_{CO} \mathcal{Y}^{\beta_{3}}_{O_{2}} Y^{\beta_{5}}_{H_{2}0} exp(-E_{2}/T)$$
(31)

and

$$Y_{H_20}(x) = \frac{m}{2} \frac{W_{H_20}}{W_{HC}} Y_{HC}(o) [\mathcal{Y}_{THC}(o) - \mathcal{Y}_{THC}(x)]$$
 (32)

Three global catalytic surface reactions are assumed to occur on the surface. They are:

(c')
$$C_n H_m + (n + \frac{m}{4}) 0_2 \rightarrow nCO_2 + \frac{m}{2} H_2 0$$
 (33)

with the corresponding surface rate J_3^{\bigstar} given by

$$J_3^* = C_3^* [C_n H_m]_s \exp(-E_3^*/RT_s^*)$$

$$(c'') C_{n}H_{m}^{(p)} + (n+\frac{m}{4})O_{2} \rightarrow nCO_{2} + \frac{m}{2}H_{2}O$$
(34)
with $J_{4}^{*} = C_{3}^{*}[C_{n}H_{m}^{(p)}]_{s} \exp(-E_{3}^{*}/RT_{s}^{*})$
(D) $CO + \frac{1}{2}O_{2} \rightarrow CO_{2}$ (35)
with $J_{5}^{*} = C_{4}^{*}[CO]_{s} \exp(-E_{4}^{*}/RT_{s}^{*})$

Other equations are similar to those in Chapter I. The steady solution is obtained from the transient combustion code as discussed in Chapter I.

II.2 Computed Results

The design parameters of a catalytic combustor include the catalytic bed length, the after-bed reaction distance, the channel hydraulic diameter, the open area percentage, the catalyst variables (catalyst type, loading density, washcoat type) and staging arrangements (graded cell, for example). The operating parameters include inlet temperature and pressure, the approaching (reference) mixture velocity, the fuel/air equivalence ratio or the adiabatic flame temperature and fuel types. An exhaustive parametric study will not be practical for this report. Instead, selected computations are presented. Since emphasis is placed on automotive gas turbine application, we are most interested in higher inlet temperature, above atmospheric pressure and high reference velocity operations with thin substrate and large open area percentage design. The total combustor length (bed plus after-bed) required to meet the emission goal (13.6 g CO/kg fuel, 1.64 g THC/kg fuel) as a function of these parameters is the main output of the model calculation.

The chosen fuel is No. 2 Diesel which was used by Anderson (1977a, 1978, 1980a). Its hydrogen-carbon atom ratio is 1.8 with 27.55% aromatics. The molecular weight is taken to be 138 for the purpose of estimating the diffusion coefficient which is assumed to vary as one over the square root of molecular weight. Other properties can be found in Anderson (1977a). The catalytic combustor used for reference calculation is the one by Matthey Bishop which was also described by Anderson (1980a) (MBI-2.5 and MBI-5.4). The values of kinetic parameters are listed in Table II.1. Expressions in Reactions A" to D are taken from Edelman and Fortune (1969), Dryer and Glassman (1973), Marteney (1979) and Kuo and Morgan (1971) respectively, all with adjusted pre-exponential

Reaction (A')	Reaction (A")	Reaction (B)	Reactions (C')(C")	Reaction (D)
$C_{0}^{\star} = 1.6 \times 10^{6}$	$C_1^{\star} = 1.8 \times 10^5$	$C_2^{\star} p^{\beta_1} = .54 \times 10^{14}$ (at 3 atm)	$C_3^* = 1.5 \times 10^4$	$C_4^{\star} = 10^5$
$\gamma_1 = 0$	$\alpha_1 = 0.3$		E 3 = 10 (kcal/mole)	$E_{4}^{\star} = 17.8$ (kcal/mole)
$Y_2 = 0.5$	$\alpha_2 = 0.5$	$\beta_2 = 1$		
$\gamma_3 = 0$	$\alpha_3 = 1$	$\beta_3 = 0.25$		
	$\alpha_4 = 1$	$\beta_4 = 0$		
E <mark>* = 3</mark> 4 (kcal/mole)	E * = 24 (kcal/mole)	$\beta_5 = 0.5$ $E_2^* = 40$ (kcal/mole)		

Table II.1 Chemical Kinetic Constants (for Chapter II)

, ·

;

4 *t*

factors. The calculated results are not very sensitive to catalytic kinetic parameters indicating mass transfer dominates the surface reactions. The choice of gas-phase hydrocarbon oxidation kinetic constants deserves some explanation, since kinetic rates for No. 2 Diesel fuel are hard to find. Ignition delay times as a function of initial temperature, however, do exist for long chain and cyclic hydrocarbons (Dryer and Glassman, 1978, Edelman, 1978), which are the ingredients of the Diesel fuel. The ignition delay time of No. 2 Diesel is assumed to be the arithmetic mean of its ingredients. Model calculations are then performed for pure gas phase reactions and the ratio of calculated hydrocarbons disappearance times as a function of initial temperature is taken to be proportional to the ratio of ignition delay times as indicated in Fig. II.1. This provides some measure for the correct temperature dependence in the gas kinetic parameters for the diesel fuel. The use of "hydrocarbon disappearance time" in this comparison reflects the difficulty of identifying ignition time in the theoretical study. The pre-exponential factor in reaction (B) is adjusted to fit the CO decay data of Anderson (1980a).

The model calculations for the catalytic combustors will be presented next. Effects of Catalytic Bed Length, Downstream Reaction Distance,

Reference Velocity and Adiabatic Flame Temperature

Figure II.2 shows the detailed profiles of two cases; one with a 5-cm bed and the other with a 2.5-cm bed. If we look at the 5-cm case first, we see when mixture enters into the reactor channel, the fuel is quickly consumed on the surface by catalytic reaction, the surface temperature starts from a low value ($T_s = 1.106$), due to the fast heat and mass transfer rates at the entrance region, and rises sharply away from the channel entrance. The gas



Fig. II.1 Comparison of temperature dependence for the computed disappearance time (nondimensional) of Diesel No. 2 fuel with Ignition Delay Data (Edelman, 1978). Reference time is taken at 1200 K.



Fig. II.2 Computed temperature and species profiles for two similar catalytic combustors with different bed lengths, $A^*/A^*_T = 0.93$ and $d^* = 1.382$ mm.

temperature begins to rise due to the heat transfer from the substrate and when T reaches 1.15, gas-phase reaction is initiated and the hydrocarbon fuel is converted to the "pyrolyzed hydrocarbon,", which is equal to the difference between \mathcal{Y}_{THC} (total hydrocarbon) and \mathcal{Y}_{HC} (original hydrocarbon). At the same time, CO, which is produced only in the gas-phase, begins to rise. The carbon monoxide in turn is consumed by both gas-phase and surface reactions to form CO₂ until the bed exit is reached. After the exit, CO is consumed by gas-phase reaction only so a slope discontinuity of the CO curve occurs at $x^* = 5$ cm. Similar discontinuity can be found in temperature and hydrocarbons, although in the latter case, it is not very apparent. If at the bed exit the gas temperature is high enough, the gas-phase reaction will continue at a high rate and adiabatic flame temperature is reached at $x^* = 12$ cm. As shown in Fig. II.2, a CO peak occurs when its production and oxidation rates become equal.

If the catalytic bed is cut short at 2.5 cm. as shown in Fig. II.2, the total hydrocarbon (\mathcal{Y}_{THC}) oxidation rate decreases suddenly at the bed exit (2.5 cm) due to the absence of surface reaction. Since the gas temperature is not yet high enough, the gas-phase reaction downstream proceeds slowyly as shown by the slower increase of gas temperature. Because a larger fraction of the hydrocarbon fuel has to be consumed in the gas-phase reaction, higher CO peak occurs and the total combustor length required to reach the emission goal is accordingly longer than the case with longer catalytic bed.

It was shown in Chapter I that the effects of catalytic bed length, after-bed distance, gas velocity and open area percentage could be correlated using reference gas residence times in the reactor bed and in the after-

bed space. For a fixed open area fraction, this correlation is simplified and can be expressed by the ratios of lengths over the reference velocity as given in Fig. II.3. For a fixed "gas residence time" in the bed (L^*/U^*_{ref}) , there is a corresponding minimum combustor residence time (L^*_{EG}/U^*_{ref}) for emissions goals to be satisfied. Although the residence time correlation is natural for gas-phase reactions, it is less obvious for surface reactions. The validity of this type of correlation is independent of the gas kinetic schemes used as illustrated here and in Chapter I where two-step kinetics and different sets of catalytic parameters were employed.

Fig. II.3 shows that as the catalytic bed length increases, the required combustor length decreases. This may lead one to choose a design which is all reactor bed without the downstream gas reaction space. However, as discussed in Chapter I, in many catalytic combustor applications (e.g., transportation engines), a fast combustor response is required. Longer reactor bed results in a longer response time due to the larger substrate thermal inertia. Therfore, certain compromises may have to be made between steady and nonsteady considerations. Take Fig. II.3, for example, one may want to choose a bed residence time of 4 milliseconds instead of 6 milliseconds. This increases the total combustor length by 15% but reduces the response time by 30%. As the inlet temperature becomes higher, as preferred by the automotive gas turbine design, the required reactor bed will be drastically reduced, leaving most reactions to occur in the gas-phase. A shorter reactor bed also decreases the pressure loss.

The importance of downstream gas-phase reactions has been recognized by Anderson as demonstrated in much of his work (Anderson, 1977b, 1980a, T'ien and Anderson, 1979). Figure II.4 compares the model calculation with his recent data (Anderson, 1980a) on CO emission index as a function of downstream



Fig. II.3 Computed "residence time" correlation: total residence time in combustor (bed plus after-bed) to reach emission goals vs. residence time in catalytic beds.



Fig. II.4 CO emission index vs. adiabatic flame temperature: comparison of model calculation with Anderson's data (1980a).

reaction distance and adiabatic flame temperature. Figure II.5 compares the minimum operating flame temperature vs. the downstream reaction distance for two bed lengths and two velocities. The comparison between the model and data is reasonably good within the range of parameters varied.

The appearance of CO peak as indicated in Fig. II.2 has been reported in previous experimental work (T'ien and Anderson, 1979, Bracco et al, 1980).

Effects of Inlet Temperature and Pressure

Figure II.6 shows the computed combustion efficiency as a function of adiabatic flame temperature for three inlet temperatures and two pressures at three different downstream locations. For a fixed pressure and adiabatic flame temperature, higher inlet temperature results in a higher efficiency at all three downstream positions. This is consistent with the experimental trend found by Anderson (1978). With higher inlet temperature, the gasphase reaction is initiated earlier which results in a higher efficiency at a fixed location. However, as more residence time is given to the mixture, the difference in efficiency diminishes as can be seen from curves at different locations in Figure II.6.

From Fig II.6, for inlet temperature of 1000 K, the combustion efficiency at 3 atm is always higher than that at 6 atm. This is because of the faster mass diffusion rate at lower pressure. For 1100 K inlet temperature, however, a portion of the 6 atm efficiency curve is higher than that of the 3 atm case as can be seen in Fig. II.6(b) and (c). This reversal phenomenon was first reported by Anderson (1978) who suggested that this was due to the competition between gas-phase and surface reactions. This model calculation



DOWNSTREAM DISTANCE (CM)

Fig. II.5 Minimum operating flame temperature (to reach emission goals) vs. downstream reaction distance: comparison of model calculation with Anderson's data (1980a).





supports this interpretation. High temperature and pressure support fast gas-phase reactions, but high pressure also retards surface reaction through slower mass diffusion rate. Taking the case of 1100 K inlet temperature and 1320 K adiabatic flame temperature for example: at the bed exit, the 6 atm case has a lower efficiency than the 3 atm case because inside the reactor channel, surface reaction dominates. At downstream distance of 5 cm, the 6 atm case still has lower efficiency. But at 10 cm downstream, the two efficiencies become equal and at 15 cm downstream 6 atm has a higher efficiency. Since only gas-phase reaction exists in the after-bed space, the efficiency reversal occurs when the gas-phase reactions overpower the upstream influence of surface reactions.

SUMMARY

A transient model of monolith catalytic combustor is presented in this work. The model assumes a quasisteady gas phase and a thermally-thin solid with the substrate thermal inertia responsible for the combustor response time delay. In the gas-phase treatment, several quasi-global chemical reactions are assumed which are capable of describing CO and unburnt hydrocarbon emissions in fuel-lean operations. By neglecting heat conduction along the flow direction in the gas and the solid in high speed approaching flows, the system of differential equations describing the combustor transient is simplified and Runge-Kutta integration scheme is applicable. The resulting computation scheme is highly efficient in computational time and is suitable for parametric calculation for both steady states and transients.

In the steady-state computation presented, the influence of selected operating and design parameters on the minimum combustor length is studied. Special attention is given to the effect of after-bed gas phase reaction space. Comparison with the data by Anderson indicates that the model is able to describe all the salient features found in the experiments including the appearance of CO peak and the efficiency reversal phenomena at high inlet temperature and pressure. Quantitative agreement with CO emission data is possible if a certain degree of rate constants adjustment is made.

The model computation also suggests that for determining the condition of achieving the steady-state emission goals, the gas residence times in the catalytic bed and in the after-bed space are the similarity parameters absorbing the influence of catalytic bed and after-bed lengths, the gas velocity and the open area fraction of the catalytic element. Likewise, the nondimensional

response time and the gas residence times are the proper similarity parameters in transient operations. A consequence of this is that the dimensional response time is proportional to the substrate crossectional area A_s^* (all other parameters fixed) and so a thin substrate is necessary for fast response.

The computed time history provides an understanding of how a catalytic combustor responds to an upstream condition change. The leading edge of the substrate is the first to respond to an upstream variation, say, a change of fuel flow rate. The leading edge is closer to the source of variation and the mass transfer rate is higher in the flow entrance region of the gas channel. As a result, the surface fuel concentration is perturbed first which, through the surface reaction, produces a surface temperature change. Heat transfer across the flow channel then causes the gas temperature to vary and this disturbance propagates downstream by convection. Since, the downstream part of the substrate always responds to the upstream substrate temperature variation, it is not difficult to see from the above description that a combustor with a longer catalytic bed residence time will take a longer time to reach a new equilibrium state.

The model calculation also shows that higher combustor inlet temperature and adiabatic flame temperature shorten the response time. Smaller channel hydraulic diameter usually results in a faster response but there are exceptions. Since the requirements for faster transient response do not always coincide with the optimal steady-state design conditions, for applications where transient characteristics are important, both steady and transient studies have to be made simultaneously so that a good compromising design can be found.

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Nomenclature

- A* crossectional area of one gas channel
- A* substrate cross-sectional area associated with one gas channel, see Fig. 1

 $A_{T}^{*} = A^{*} + A_{S}^{*}$, total crossectional area of one cell unit

$$B_{0} = \frac{L^{*}}{U^{*}(0)} \frac{1}{\rho^{*}(0)} \frac{1}{Y_{HC}(0)} C_{0}^{*} p^{*}(0) \rho^{*}(0) \tau^{*}(0) Y_{HC}^{2} T^{*}(0) Y_{HC}^{2}(0) W_{HC}^{(1-Y_{2})}$$

$$\mathbf{B}_{1} = \frac{\mathbf{L}^{*}}{\mathbf{U}^{*}(0,0)} \frac{1}{\rho^{*}(0,0)} \frac{1}{\mathbf{Y}_{HC}(0,0)} \frac{1}{\mathbf{Y}_{HC}(0,0)} \mathbf{C}_{1}^{*} \frac{\rho^{*}(0,0)}{\rho^{*}(0,0)} \frac{\rho^{*}(0,0)}{\rho^{*}(0,0)} \mathbf{T}_{1}^{*}(0,0) \mathbf{Y}_{HC}^{\alpha_{2}}(0,0) \mathbf{Y}_{02}^{\alpha_{3}}(0,0) \mathbf{Y}_{02}^{\alpha_{3}}(0,0) \mathbf{Y}_{02}^{\alpha_{3}}(0,0) \mathbf{Y}_{02}^{\alpha_{3}}(0,0)$$

$${}^{B}_{2} = \frac{L^{*}}{U^{*}(0,0)} \frac{1}{\rho^{*}(0,0)} \frac{1}{Y_{HC}^{(0,0)}} \frac{C^{*}_{2}}{Y_{HC}^{(0,0)}} \rho^{*}(0,0)^{\beta_{1}} \rho^{*}(0,0)^{(\beta_{2}+\beta_{3}+\beta_{5})} T^{*}(0,0) Y_{HC}^{\beta_{2}}(0,0)$$

× $Y_{02}^{\beta_{3}}(0,0) W_{C0}^{(1-\beta_{2})} W_{02}^{-\beta_{3}} W_{H_{2}0}^{-\beta_{5}}$

$$B_3 = C_3^* d^* / \alpha^* W_{HC} Nu_*$$

$$B_4 = C_4^* d^* / \alpha^* W_{CO} Nu_{\bullet}$$

$$C_n = n W_{CO} / W_{HC}$$

C^{*}₅ specific heat of substrate

C* gas specific heat

d* catalytic monolith combustor channel equivalent diameter

d* half thickness of the substrate

E^{*} activation energy

 J_3^* mass consumption rate of hydrocarbons per unit surface area through reaction (C)

 J_4^{\bigstar} mass consumption rate of CO per unit surface area through reaction (D)

$$J_{Di} = J_{H} (Le_{i})^{2/3}$$

$$J_{\mu} = 4 Nu_{(\alpha^{*}(0,0)/d^{*})}(L^{*}/u^{*}(0,0)(k^{*}/k^{*}(0,0)))$$

- k* heat conductivity of the gas
- L* catalytic bed length
- L_{as}^{\star} length of after-bed downstream gas phase reaction space
- L* total combustor length (catalytic reactor length plus after-bed downstream gas reaction distance)
- L^{*} EG total combustor length (catalytic bed plus after-bed space) required to meet emission goals
- Le, Lewis number for hydrocarbon gas in air
- Le₂ Lewis number for CO in air
- m number of hydrogen atom in C_nH_m.
- n number of carbon atom in C_n_m
- Nu____ Nusselt number at x
- Nu_ Nusselt number for fully developed flow
- p = p*/p*(0,0), nondimensional pressure
- $q_1 = q_1^*/C_p^*T^*(0,0)$, nondimensional heat of combustion per unit mass of $C_n H_m$ in reaction A
- q₂ = q^{*}₂/C^{*}_pT*(0,0), nondimensional heat of combustion per unit mass of CO in reaction B
- q₃ = q^{*}₃/C^{*}T^{*}(0,0), nondimensional heat of combustion per unit mass of C_nH_min reaction C
- $q_4 = q_4^*/C_7^*T^*(0,0)$, nondimensional heat of combustion per unit mass of CO in reaction D, same as q_2

r see Eq. (17)

- S* circumferential length of one channel crossection (= πd*)
- $T = T^*/T^*(0,0)$, nondimensional gas temperature

 $T_{s} = T_{s}^{*}/T^{*}(0,0)$, nondimensional substrate temperature

- T* adiabatic flame temperature
- T* combustor inlet temperature
- $t = t*/\tau*$, nondimensional time
- $t_b^* = (L^*/U_{ref}^*)$ (A*/A*), reference gas residence time in the catalytic combustor bed
- $t_{as}^{*} = L_{as}^{*}/U_{ref}^{*}$, reference gas residence time in the after-bed space
- t^{*}_{ch} characteristic transient response time ($\eta_7 \rightarrow 80\%$)
- t_{EG}^{\star} Minimum total combustor gas residence time $(t_b^{\star} + t_{as}^{\star})$ needed to reach emission goals

u*/u*(0,0), nondimensional flow velocity

- "ref reference gas velocity, measured at upstream of catalytic bed
- W, molecular weight of species i
- w, chemical reaction rate
- x = x*/L*, nondimensional axial distance measured from catalytic bed entrance to downstream
- Y mass fraction of species i
- $y_{\rm CO} = y_{\rm CO}/y_{\rm HC}^{(0,0)}$

u

- $y_{02} = y_{02}/y_{02}(0,0)$
- $\mathcal{G}_{HC} = \mathbf{Y}_{HC} / \mathbf{Y}_{HC}(0,0)$
- $\mathcal{G}_{HC}^{(p)} = \gamma_{HC}^{(p)} / \gamma_{HC}^{(0)}$

 $\mathcal{Y}_{\text{THC}} = \mathcal{Y}_{\text{HC}} + \mathcal{Y}_{\text{HC}}^{(p)}$

α*	thermal diffusivity of gas
^α i	rate exponent, see Eqs (1) and (23)
ß	rate constant, see Eq. (22)
$^{\beta}$ i	rate exponent, see Eqs (2) and (24)
Υ _i	rate exponent, see Eq (22)
p*	gas density
₽ ★ s	substrate density
τ*	characteristic substrate heat-up time, see Eq. (16)
η _{CB}	carbon-balanced efficiency
ⁿ T	combustion efficiency based on temperature difference

[] concentration, g-mole/c.c.

Subscript s

surface

Superscript

dimensional quantity

APPENDIX I

LISTING OF COMPUTER PROGRAMS FOR CHAPTER I

The numerical computations were performed using an interactive VAX -11/780 computer. FORTRAN language was used in writing the main program (e.g. IGNITION. FOR; 17) and the data program (e.g. BLOCK. FOR; 31). They were then compiled to form the objective programmes (e.g. IGNITION. OBJ; ** and BLOCK. OBJ; **). The two objective programmes were then linked together to form an executable program (e.g. IGNITION. EXE; 10). In performing a calculation, only the executable is needed.

The BLOCK program stores most of the data which are not likely to be changed from one calculation to the other, while the most frequently varied input data will be read in in each computer run. If the data stored in the BLOCK program are to be changed, the FORTRAN BLOCK program has to be modified (edited), re-compiled and linked with the main program to form a new executable program.

In the following, the FORTRAN programmes (IGNITION. FOR; 17 and BLOCK. FOR; 31) are listed together with a sample calculation showing input and output data.

The IGNITION program is presently set up to perform the start-up transient. With minor modification, it can read in any prescribed initial condition and to calculate the corresponding unsteady response. Given enough computational time, the IGNITION program will also result in the steady-state profiles.

More detailed instrictions (for IGNITION. FOR; 17):

- The number of grid points used should be less than 300. If more than 300 points are needed, line 600 has to be changed.
- 2. Read-in data (lines 10900 12300): XPHI = ϕ = fuel-air equivalence ratio

TI = T (0,0) = upstream, initial temperature (K) XPI = P(0,0) = upstream, initial pressure (Atm.) UIUS = U_{rof} = upstream (outside of the bed) initial gas velocity (m/s) XLS = reference length (m), suggested to use 0.1 m (10 cm) DS = d* = channel hydraulic diameter (m) $AOAS = A*/A*_{a} = open-to-close area ratio$ D_X = nondimensional step size in X (space). It is nondimensionalized by XLS. So if XLS = 0.1 m and DX = 0.01, dimensional step size is 0.001 m or 1 mm. DT = nondimensional time step size TMAX = maximum nondimensional time allowed for computation IPRINT = time interval for printing out transient profiles. For example, if DT = .01 and IPRINT = 25, Output will be printed out every t = 0.25. N = number of grid points (in X direction) in the catalytic bed. The nondimensional catalytic bed length is given by $(N-1) \times (DX)$. For example, if N = 51, DX = .01, then the nondimensional bed length is equal to 0.5. If XLS = .1 m, then the dimensional bed length is .05 m (5 cm). NAS = number of grid points (in X-direction) in the downstream after-bod space. The length of the downstream distance can be calculated in a similar manner as in the above. C1, C2, C3, C4 = C_1 , C_2 , C_3 , C_4 (See Table I.2) IT = interval in X for printout. For example, if DX = 0.01 IT = 5, the printout will be at X = 0, 0.05, 0.1, 0.15, ...etc. 3. Those data which are not read in will be defined by the BLOCK program.

4. Some Symbol Definitions:

PR = Prandtl number of air

REY = Reynolds number based on channel diameter and upstream velocity

$$ES, S = E_1^*, E_2^*, E_3^*, E_4^*$$
 (See Table I.2)

 $E,S = E_1, E_2, E_3, E_4$

 $C_{1}S = C_{1}^{*}, C_{2}^{*}, C_{3}^{*}, C_{4}^{*}$ (See Table 1.2)

GC = gas constant (joule/kg K)

XLEW1 = Lewis number for C_{n-m}^{H} in mixture

XLEW2 = Lewis number for CO in mixture

ROI = initial, upstream (I.U.) gas density (kg/m^3)

XKI = I.U. gas heat conductivity (cal/m sec K)

CP = specific heat of gas at constant pressure (cal/kg K) ALPHA = I.U. gas diffusivity (m²/sec)

 $TS = T_{s}$ YFS = $\mathbf{y}_{HC,s}$

YCS = $\mathbf{y}_{CO,s}$ YF = \mathbf{y}_{HC} YO = \mathbf{y}_{O_2} TMAX = Ma

TMAX = Maximum time for computation

 $YC = \mathbf{y}_{CO}$

5. Output data (lines 40300 to 42200)

The following quantities are printed out as a function of axial distance and time:

Nondimensional gas and surface temperatures, Emission indices (x1000) of hydrocarbon fuel and CO (both in the gas phase and on the surface) and carbon-balanced and thermal efficiencies.

IGNITIO	N.FOR	;17	12	-APK	-1981	23:	29:54	.94	PAGE 1
100 200 300 300	COCOC	**************** ******CATALYIIC *************	*** CDM ***	**** EUST ****	**** OR ST ****	**** APT- ****	***** UP 1F ****	*********** ANSIENT**** *********	******
000 700 900 1000 1100 1200		PARAMETER NX=300 COMMON /ACTI/ES1, COMMON /ACTI/ES1, COMMON /TH1/XN, X COMMON /TH1/XN, X COMMON /TH2/CF, COMMON /IN2S1/YC1 COMMON /IN1/TS,}F	ES2 HC, GC, S,Y	,ES3 WCD G1S, XPR CI CS	,ES4 , wO2 Q25	, wH	20		
1400 1500 1600 1700		DIMENSION YF(NX), DIMENSION XKY(4), DIMENSION XJH(NX DIMENSION EFF(NX)	т(ХК), R	NX) 1(4) XJD1 NK(N	TS(N , XKT (NX) X),EF	X), S(4) XJD FT(N	YD(N) 2 (NX) 2 (NX) X)	(), YFS(NX) 2(4), YC(NX)), YH2C(NX)	RD(NX), U(NX) , YCS(NX) ,XP(NX)
1900 2000 2100 2200	ر ب	7 FORMAT (4E20.10) 8 FORMAT (14) 9 FORMAT (140,'SCL1 0 FORMAT (6F12.6) 1 FORMAT (6F12.6)	D T	EFPE	RATUR	E (1	5)'/)		
2400 2500 2600 2700 2750 2800	111111	2 FORMAT (//) 3 FORMAT (//) 4 FORMAT (1H0,'GAS 5 FORMAT (1H0,'CNHM 5 FORMAT (1H0,'YG(UP 1,5X,'EQU1 RATIG 7 FORMAT (1H0,'GI='	TEM MA ST FI	FERA SS F INI) 5.9/	TURE RACTI =',F /)	(T)' GN (10.6	110N YF) +/ ,5X, -	(IFS) () () () () () () () () () () () () () ()=',F10.7 F15 7.5% !FT='.
29000 3000 3100 3200 3500	1 1 2 2 2	1715.7//) 8 FORMAT (1HO,'N=', 9 FORMAT (1HO,'INIT 0 FORMAT (1HO,'INIT 1 FORMAT (1HO,'IMAX 2 FORMAT (1HO,'TIMAX	I10 IAL DF	CON GRID	DITIC 5.8) POIN 8.5X	NS (TS)', ,211(,/) ,/) ,/)	13.7,5%, 51- ,
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5500 5600 5650	4	FORMAT (1H0, GC, 9 FORMAT (1H0, RCI,	XLE XK	W1, 1, C	XLEW2 P, AL	Х1 ¢на	51,08 =1,31	5 =',6F12.6/ F12.6,E12.5/)
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7400 7500 7600		*************INPUT *********CHEMICA	PA L P	RAME ARAM	TERS.	**** ****	*****	****	
7700 7800 7900 8000 8100	υυυυ	CHEMICAL REACTION CHEMICAL REACTION CHEMICAL REACTION	1, 2, 3,	CN CD CNH	HM + + 02 M + 0	C2 = = CC 2 =		H2C (GAS P As Phase) H2C (Solid	HASE) Surface)

CHEMICAL REACTION 4, CO +O2 = CO2 (SOLID SURFACE) CO CO CO CO CO CO CO CO CO CO	,ETC. TICN 1, E1 CNHM) CC) CNHM)
00 C 00 C 00 C 00 C 00 C 00 C 1 = ACTIVATION ENERGY FOR REACTION 1 (CAL/MOLE) 00 C 01 = PREEXPONENTIAL FACTOF (DIMENSIONAL) FOR REACTION 01 C 020 C 031 = MGLECULAR WEIGHTS FOR CNHM, CU 040 C 050 C 051 = HEAT OF COMBUSTION FOR REACTION 1 (CAL/G OF 050 C 052 = HEAT OF COMBUSTION FOR REACTION 3 (CAL/G OF 050 C 050 C <td< td=""><td>,ETC. TICN 1, E1 CNHM) CU) CNHM)</td></td<>	,ETC. TICN 1, E1 CNHM) CU) CNHM)
D0 C C1 = PREEXPONENTIAL FACTOF (DIMENSIGNAL) FOP REAC D0 C WHC, WCU = MOLECULAR WEIGHTS FOR CNHM, CU D0 C G1 = HEAT OF CCMBUSTION FOR REACTION 1 (CAL/G OF D0 C G2S = HEAT OF CCMBUSTION FOR REACTION 2 (CAL/G OF D0 C GS = HEAT OF CCMBUSTION FOR REACTION 3 (CAL/G OF D0 C SS = HEAT OF CCMBUSTION FOR REACTION 3 (CAL/G OF D0 C XN = THE NUMBER OF CAFEONS IN CNHM D0 C XM = THE NUMBER OF HYDROGEN IN CNHM D0 C XM = THE NUMBER OF HYDROGEN IN CNHM D0 C C D0 </td <td>TICN 1, E1 CNHM) CC) CNHM)</td>	TICN 1, E1 CNHM) CC) CNHM)
20 C Q1S = HEAT OF CCMBUSTION FOR REACTION 1 (CAL/G OF 00 C Q2S = HEAT OF CCMBUSTION FOR REACTION 2 (CAL/G OF 00 C QS = HEAT OF COMBUSTION FOR REACTION 3 (CAL/G OF 00 C $XN = THE NUMBER OF CAFEONS IN CNHM 00 C XM = THE NUMBER OF HYDROGEN IN CNHM$	CNHM) CC) CNHM)
00 C GS = HEAT OF COMBUSTION FOR REACTION 3 (CAL/G OF 00 00 C XN = THE NUMBER OF CAREONS IN CNHM 00 C XM = THE NUMBER OF HYDROGEN IN CNHM 00 C 00 C	CNHM)
00 C XN = THE NUMBER OF CAREGNS IN CNHM 00 C XM = THE NUMBER OF HYDROGEN IN CNHM 00 C 00 C 00 C 00 C	
$UU \qquad US = UIS + UZS = XN + W(L/WH)$	
00 C 00 C +******* CATALYTIC BED PARAMETERS ***************	
DO C XLS = BED LENGIH (M)	
00 C DS = CHANNEL HYDRAULIC DIAMETER (M) 00 C SOA = CIRCUMFERENTIAL LENGTH DIVIDEC BY CHANNEL A	REA,=4/DS(
DO C ADAS = RATIO OF UPEN TO CLOSED AREAS DO C CSS = HEAT CAPACITY OF SUBSTRATE (CAL/KG/K)	
00 C RDS = DENSITY OF SUBSTRATE (KG/M3) 00 C	
00 C 00	
DO READ (5,10) 11 DO REAU (5,10) XFI	
00 REAC (5,10) UIUS 00 REAL (5,10) XLS	
00 REAC (5,10) DS 00 REAC (5,10) AGAS	
00 REAL (5,10) DX 00 READ (5,10) D1	
00 - REAC (5,10) TMAX 00 - READ (5,8) IPRINT	
00 READ (5,8) N 00 READ (5,8) NAS	
00 REAL (5,*) C1,C2,C3,C4 00 REAL (5,8) IT	
00 SGA = 4.0/DS 00 CN = XN*WCG/WHC	
DO NPNAS = N + NAS DO C	
DO C	• •
DO C UI = I.U. VELOCITY (M/S) DO C XPI = I.U. PRESSURE (ATM.)	
DO C RCI = I.U. GAS DENSITY (KG/M3) DO C YFI = I.U. MASS FRACTION OF HC	
DO C YCI = I.U. MASS FRACTION OF CC DO C YOI = 1.U. MASS FRACTION OF C2	
00 C XPHI = UPSTREAM HC/02 EQUIVALENCE RATIO 00 C	
00 C 00 UI = UIUS*(1.0+, UAS)/ADAS	
00 XMU = 0.000001458+SQRT(TI)/(1.0+110.4/TI) 00 YFI = WHC*X ⁺ HI/29./((XN+XM/4.)*4.76+XPHI)	
DO XLEW1 = 1.5* SQRT (29./WHC) DO XLEW2 = 1.5* SQRT (29./WCC)	
00 XLE123=XLEW1**.667 00 XLE223=XLEW2**.667	
DO C XLOU = XLS/UI	•
00 C ************* GAS PROPERTY ************************************	
00 C 00 C	
DO C CP = SPECIFIC HEAT OF GAS (CAL/KG/K) DO C XKI = HEAT CONDUCTIVITY OF GAS AT I.U. (CAL/M/SEC	:/K)
DO C ALPHA = I.U. GAS THERMAL DIFFUSIVITY (M2/SEC) DO C GC=GAS CONSTANT FOR GAS MIXTURE (JOULE/KG/K)	
JO C XLEWI = LEWIS NUMBER FOR CNHM IN MIXTURE JO C XLEW2 = LEWIS NUMBER FOR CO IN MIXTURE	
50 C	
00 XKI#.003+.000013∓TI 00 RDI.# XPI/GC/TI	

IGNITIC	DN.FOP;1	7 12-APR-1981 23:29	9:54,94	PAGE
16300	с	REY = UIUS*DS*PCI/XMU		
16500	ç	TO DESCRIBE CHEMICAL REACTION RATE	EXPRESSIONS	
16800	Č	D(CNHM)/UT \$55 XP**.3*T*SORT(YHC)*)	(O2/EXP(E1/RT)	
17000	č	RFACTION 2 (GAS PHASE) D(CC)/DT \$\$\$ YCC+SQRT(YC2)+SCRT(YH2	2G)/EXP(E2/R1)	
17200	C .	REACTION 3 (SOLID SURFACE)		
17500 17600	Č	REACTION 4 (SOLID SURFACE)		
17700	cc	D(CC)/DT \$\$\$ YCCS/EXF(E4/RT)		
18000	č	BISS = C1+SORT(NHC)/NC2/SORT(1000.	h	
18200 18300 18400	_	BIS = BISS*XPI**, 3*RCI*SGRT(RCI)*T B1 = XLUU*BIS/RCI/YFI SS%D2=SGRT(SGRT(WD2))	Í*SQRT(YF1)*YO1	
18500	С	B2SS = C2/1000.**.75/SSW02/SORT(WH	20)	
18900		SSYCI=SORT(SORT(YOI)) B2S=B2SS*ROI*RCI/SSRCI*YFI*SSYOI		
19000	с	B2 = B2S*XLOU/RC1/YFI		
19300		B4 = C4+DS/ALPHA/WHC/3.6601 = G1S/CP/T)		
19500 19600		$\vec{0} = \vec{0} \vec{2} \vec{5} / \vec{C} \vec{P} / \vec{1} \vec{1}$ $\vec{0} = \vec{0} \vec{5} / \vec{C} \vec{P} / \vec{1} \vec{1}$		
19700		TABDM = TABTI FABDM = TABTI F1 = FS1/T/1 997		
20000		$E_{2} = E_{2}^{2} (11/1.947)$ $E_{3} = E_{3}^{2} (11/1.987)$		
20200	-	E4 = ES4/TI/1.987 AD2UL = (ALPHA/CS/DS)/(UI/XLS)		
20400	C C	WRITE (6.21) N NAS		
20700		WRITE (6,20) TMAX WRITE (6,15) YCI, YFI, XPHI		
20900		WRITE (6,71) UTUS WRITE (6,72) UT		
21200 21200 21300		WRITE (6,402)TI,XPR,REY, AD2UL, A0/ WRITE (6,402)TI,XPR,REY, AD2UL, A0/	AS	
21400 21500		WRITE (6,410) TABDM WRITE (6,403)ES1, ES2, ES3, ES4		
21600		WRITE (6,47)E1, E2, E3, E4 WRITE (6,35) C1, C2, C3, C4 WDITE (6,48) CC YTELL YTEL2 YTE	DS	
21900 22000		WRITE (6,49) RCI, XKI, CÊ, ALFHÂ WRITE (6,401) XFI		
22100		WRITE (6,43) XLCU WRITE (6,44) B1S, B2S		
22400		WRITE (6,19) WRITE (6,19) WRITE (6,24) (TS (I), I=1.N.IT)		
22600		WRITE (6,25) WRITE (6,24) (YFS(I), I=1,N,IT)		
22800	c	WRITE (6,33) WRITE (6,24)(YCS(I), I=1,N,IT)		
23100	Č	*******		
23300	С	************** UPSTREAM CONDITIONS *: YF(1) = 1.	********	
23600		$Y_{0}(1) = 1$		
23800 23900		$\vec{RO(1)} = 1$. $\vec{XP(1)} = 1.0$		
24000 24100	ç	U(1) = 1.		
24300	č	*****	τ τ τ Φ Φ.Φ Φ Τ Τ Τ Τ	

IGNITION.	FOR	; 1	7											1.2	-1	ΝP	R-	- 1	9	61		23	J:	29	•:	54	١.	9	4						PA	GE	: 4	ł
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26200 C 26300 C			YF	RE RE	FI C1	JE.	L UN	M	AS JS	S IN	Fi IG	R A F	C O	II JR	01 71	() -	DF	N N R D	D	Ţ	R	(1) U N	E IG	M F E =	E	R / U 1	T	U	PES	() ()	HE	E N E M	ED	0	N S	5 T R	EA	M
26400 Č			1	15	3	[H]	Ê	11	ND	E)		FO	R	D	C	N	Ŝ1	[R	Ē	M		DI	S	ĪA	N	CE	E	X		•		-	-					
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N.FOR;17 12-APR-1981 23:29:54.94 PAGE 5
119 CCMIINUE
YH2C = MASS FRACTION OF H2C
YH2C(I) = .5*XM*WH2C*YFI* (YF(1)-YF(1))/*HC
If(Y+F,LT.0.0) YFE=0.0
w1 = H1* XP(I)**.3*PC(I)*SORT(RO(I))*T(I)*SURT(YFF)*YC(I)
SSYC=SUHI(SORT(YU(I)))
R0175=RO(I)*L+C(I)/SURI(SORT(KC(I)))
w2=E2*RO(75*YCC*SSYC*SOPT(YH2C(I))/EXP(E2/TT)
XKY(L) = -XJD1(I) *(YFF- YFS(I))/DMASS -*M1/DMASS
XKC(L) = -XJD2(I) *(YCC+YCS(I))/DMASS +C1*#1/UMASS+02*YFI*#2/DMASS
120 CONTINUE
YF(I+1) =YF(I) +DX*(XKY (1)/6. +XKY (2)/3. +XKY (3)/3.+XKY (4)/6.)
IF (YF(I+1)=YF(I) +DX*(XKY (1)/6. +XKY (2)/3. +XKT (3)/3.+XKI (4)/6.)
IF (YC(I+1)=YC(I)+EX*(XKC(I)/6.+XKT (2)/3. +XKT (3)/3.+XKI (4)/6.)
IF (YC(I+1)=YC(I)+EX*(XKC(I)/6.+XKT (2)/3. +XKT (3)/3.+XKI (4)/6.)
IF (T(I+1)=T(I)+DX*(XKT (1)/6.+XKT (2)/3. +XKT (3)/3.+XKI (4)/6.)
IF (T(I+1)=XP(I)/T(I+1)=0.0
T (I+1)=XP(I)/T(I+1)U(I+1)+ADD
XP(I+1)=XP(I)/T(I+1)U(I+1)+ADD
XP(I+1)=XP(I)/T(I+1)U(I+1)+ADD
XP(I+1)=XP(I)/T(I+1)U(I+1)=XP(I)/T(I+1)U(I+1)=XP(I)/T(I+1)U(I+1)U(I+1)U(I+1)=XP(I)/T(I+1)U(I+1)U(I+1)U(I+1)=XP(I)/T(I+1)U(I+1)U(I+1)U(I+1)U(I+1)=XP(I)/T(I+1)U(I+1 IGNITION.FOR;17 32400 32500 32650 32650 32700 32800 33200 333000 333000 33300 33300 33300 33400 33500 33600 33700 END OF ONE RUNGE-KUTTA INTEGRATION CYCLE FOR QUASI-STEADY GAS PHASE EQUATIONS ******* INTEGRATION OF UNSTEADY SOLIC HEAT TRANSFER EQUATION FOR TS (SOLID TEMPERATURE) IN TIME USING FOURTH-ORDER RUNGE-KUTTA SCHEME DC 301 J =1,N MJ=1 DC 320 LJ=1,4 IF (MJ.NE.1) GC TO 302 ISS = TS(J) MJ=KJ+1 GO TO 319 302 CONTINUE IF(MJ.NE.2) GO TO 303 ISS = TS(J) + DT*XKTS(1)/2. MJ=MJ+1 GO TO 319 303 CONTINUE ISS=TS(J)+DT*XKTS(2) /2. MJ=MJ+1 GO TO 319 304 CONTINUE XKTS(LJ)=RNK(J)*(T(J)-TSS) +E3*YFI*O*RO(J)*YFS(J)/EXP(E3/TSS) 1 +E4*YFI*U2*RO(J)*YCS(J)/EXP(E4/TSS) 320 CONTINUE TS(J) =T5(J) + DT*(XKTS(1)/6.+XKTS(2)/3.+XKTS(3)/3.+XKTS(4)/6.) YFS(J) = TF(J) /(1.+E3*RO(J)/XLE123/RNK(J)/EXP(E4/TS(J))) 301 CONTINUE END OF ONE TIME INTEGRATION STEP DC 301 J =1,N 36600 36700 36800 36900 **3**7600 37100 37200 37300 37400 000000 39000 39100 39200 39300 39400 39500 END OF ONE TIME INTEGRATION STEP LOOP FOR MARCHING FORWARD IN TIME 39600 39700 39800 39900 40000 С KOUNT = KOUNT+1 KOU = KOUNT-1 XKOUNT = KOUNT TIME=(XKOUNT-1,)*DT IF (TIME.GT. TMAX) GC TO 999 IF (MOD(KOU,IPFINT).NE.0) GD TO 200 40100

IGNITION.FOR;1	7 1	L2-APR-1981 23:29:54.94	FAGE 6
40300	WRITE(6,22) TIME ,	KCUNT	
40500	WRITE(6,24) (T_ (I) WRITE (6,9)),1=1,NFNAS,1T)	
40700	WRITE(6,24) (TS (1) WRITE(6,14)), I=1,N,IT)	
40900 41000	WRI1E(6,24) (YF(1) WRITE(6,41)),I=1,NPNAS,IT)	
41100 41200	WRITE(6,24) (YC(I) WRITE(6,11)),I=1,NPNAS,IT)	
41300	WRITE(6,24) (YFS() WRITE(6,42)	[),I=1,N,IT)	
41500 41600	WRITE(6,24) (YCS() WRITE(6,26)	l),I=1,N,IT)	
41700 41800	WRITE(6,24) (XJH(1) WRITE(6,73)),I=1,N,IT)	
41900 42000	WRITE(6,24) (EFF(1 WRITE(6,74)	l),I=1,NPNAS,IT)	
42100 42200	WRITE(6,24) (EFFT(WRITE(6,23)	(I), I=1, NPNAS, 1T)	
42300 42400 C	GO 10 200		
42500 C 42600 C	END OF WHOLE INTEGR	\A'IION LOOP ***********	******
42700 C 42800 601	CONTINUE		
43000	WRITE (6,404) WRITE(6,405)		
43200 43300	CUNTINUE STCP END		

100	BLOCK DATA
300	DIMENSION TS(300), YES(300), YCS(300)
600	COMMON /MOLEGY NHC, WCC, WC2, NH20
800	COMMEN /TH2/ CP, GC, XPR
1100	COMMON /INI/TS,YFS,YCS
1400	DATA ES1, ES2, ES3, ES4/24000., 40000., 10000., 17000./
1600	DATA XN, XM, Q1S, C2S/3, 8, 6491000, 2415000./ DATA CP. GC. XDN/265, 2, 8, 6491000, 2415000./
1800	DATA YCI, YOI/0.,0.23/ DATA TS.YES.YCS/300*1300*0300*0./
2100	END
Example calculation:



TMAX = 1.00100005

YD(UPST,INI) = 0.230000 YF(UPST,INI)= 0.0113888 EQUI RATID = 0.18000 0007

APPROACH YEL = 10.0000

CHANNEL VEL (1.0.) = 15.00000

01= 22.7754402 02= 8.4736843 DX= 0.0100000 DT= 0. 0100000

TI, PR, REY, AD2UL, ADAS = 1000.00000 0.70000 408.31238 0.13797 2 .00000

ADIABATIC FLAME TEMP = 1.443622

ADIAB FLAME TEMP(K) = 1443.62195

ES+S≠ 0.24000000E+05 0.4000000E+05 0.10000000E+05 0.178000 00E+05

E,S = 12.078510 20.130850 5.032712 8.958228

C:S = 0.150000000E+06 0.7100000371E+14 0.2500000000E+04 0.100000 0000E+06

5C, XLEW1, XLEW2, XLS, DS = 0.002831 1.217766 1.526551 0.100000 0 .001600

RDI, XKI, CP, ALPHA = 1.059621 0.016000 285.000000 0.52982E-04

XPI = 3.00000

L/U = 0.00666667

B1S, B2S, = 0.36601332E+05 0.34534906E+09

R1, R2, R3, R4 = 0.20219805E+05 0.19078240E+09 0.46881531E+03 0.29468391E+05

INITIAL CONDITIONS (TS)

0.10000E+01 0.10000E+01 0.10000E+01 0.10000E+01 0.10000E+01 0.10000E+01 0.10000E+01 0.10000E+01 0.10000E+01

INITIAL CONDITIONS (YES)

0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E +00 0.00000E+00 0.00000E+00 0.00000E+00

INITIAL CONDITIONS (YCS)

0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00

TIME= 0.20000000 KOUNT= 21

GAS TEMPERATURE (T)

0.10000E+01 0.10279E+01 0.10351E+01 0.10393E+01 0.10420E+01 0.10437E+01 0.10447E +01 0.10451E+01 0.10451E+01 0.10447E+01 0.10462E+01 0.10481E+01 0.10501E+01 0.10521E+01 0.10541E+01 0.10562E+01 0.10583E +01 0.10604E+01 0.10625E+01 0.10647E+01 0.10669E+01 0.10691E+01 0.10714E+01 0.10737E+01 0.10760E+01 0.10784E+01

SOLID TEMPERATURE (TS)

0.11390E+01 0.10786E+01 0.10631E+01 0.10542E+01 0.10477E+01 0.10427E+01 0.10384E +01 0.10349E+01 0.10321E+01 0.10295E+01

CNHM MASS FRACTION (YF)

0.10000E+01 0.82811E+00 0.72848E+00 0.64850E+00 0.58054E+00 0.52141E+00 0.46932E +00 0.42307E+00 0.38150E+00 0.34400E+00 0.33115E+00 0.32383E+00 0.31646E+00 0.30902E+00 0.30153E+00 0.29398E+00 0.28638E +00 0.27871E+00 0.27100E+00 0.26324E+00 0.25542E+00 0.24756E+00 0.23966E+00 0.23171E+00 0.22373E+00 0.21571E+00

YCO =

0.00000E+00 0.11484E-01 0.22136E-01 0.31487E-01 0.39522E-01 0.46303E-01 0.51923E -01 0.56483E+01 0.60031E-01 0.62661E-01 0.73834E-01 0.87281E-01 0.10073E+00 0.11418E+00 0.12762E+00 0.14103E+00 0.15440E +00 0.16772E+00 0.18098E+00 0.19417E+00 0.20726E+00 0.22024E+00 0.23309E+00 0.24579E+00 0.25833E+00 0.27068E+00

SURFACE HC MASS FRACTION (YFS)

0.53851E+00 0.24678E+00 0.20088E+00 0.17326E+00 0.15271E+00 0.13598E+00 0.12175E +00 0.10985E+00 0.99795E+01 0.90590E-01

YOD (SURFACE) =

0.00000E+00 0.26429E-02 0.48789E-02 0.68753E-02 0.86412E-02 0.10176E-01 0.11484E -01 0.12628E-01 0.13636E-01 0.14440E-01

 $HL \times$

0.11677E+02 0.32271E+01 0.26856E+01 0.24599E+01 0.23328E+01 0.22489E+01 0.21875E. +01 0.21536E+01 0.21464E+01 0.21403E+01

EFFICIENCY (CARBON-BALANCE)

0.00000E+00 0.16939E+00 0.26670E+00 0.34465E+00 0.41086E+00 0.46852E+00 0.51939E +00 0.56464E+00 0.60544E+00 0.64237E+00 0.65279E+00 0.65718E+00 0.66163E+00 0.66614E+00 0.67071E+00 0.67534E+00 0.68004E +00 0.68480E+00 0.68963E+00 0.69452E+00 0.69949E+00 0.70453E+00 0.70964E+00 0.71482E+00 0.72008E+00 0.72541E+00

EFFICIENCY (TEMPERATURE)

0.00000E+00 0.62838E-01 0.79086E-01 0.88607E-01 0.94669E-01 0.98477E-01 0.10066E +00 0.10160E+00 0.10161E+00 0.10085E+00 0.10409E+00 0.10848E+00 0.11292E+00 0.11743E+00 0.12200E+00 0.12663E+00 0.13133E +00 0.13609E+00 0.14092E+00 0.14582E+00 0.15079E+00 0.15582E+00 0.16093E+00 0.16611E+00 0.17137E+00 0.17670E+00

TIME= 0.40000001 KDUNT=

GAS TEMPERATURE (T)

0.10000E+01 0.10491E+01 0.10635E+01 0.10720E+01 0.10776E+01 0.10811E+01 0.10833E +01 0.10843E+01 0.10845E+01 0.10841E+01 0.10861E+01 0.10889E+01 0.10917E+01 0.10945E+01 0.10975E+01 0.11004E+01 0.11035E +01 0.11065E+01 0.11097E+01 0.11129E+01 0.11161E+01 0.11194E+01 0.11228E+01 0.11262E+01 0.11297E+01 0.11332E+01

41

SOLID TEMPERATURE (TS)

0.12213E+01 0.11518E+01 0.11246E+01 0.11083E+01 0.10964E+01 0.10869E+01 0.10789E +01 0.10722E+01 0.10667E+01 0.10617E+01

CNHM MASS FRACTION (YF)

0.10000E+01 0.80758E+00 0.70137E+00 0.61707E+00 0.54601E+00 0.48467E+00 0.43108E +00 0.38389E+00 0.34185E+00 0.30433E+00 0.28925E+00 0.27908E+00 0.26883E+00 0.25851E+00 0.24813E+00 0.23769E+00 0.22719E +00 0.21666E+00 0.20609E+00 0.19550E+00 0.18489E+00 0.17429E+00 0.16370E+00 0.15314E+00 0.14264E+00 0.13221E+00

YCD =

0.00000E+00 0.13222E-01 0.26772E-01 0.39234E-01 0.50220E-01 0.59608E-01 0.67399E -01 0.73657E-01 0.78404E-01 0.81781E-01 0.96870E-01 0.11495E+00 0.13288E+00 0.15060E+00 0.16809E+00 0.18529E+00 0.20218E +00 0.21869E+00 0.23479E+00 0.25040E+00 0.26548E+00 0.27995E+00 0.29376E+00 0.30682E+00 0.31907E+00 0.33043E+00

SURFACE HC MASS FRACTION (YES)

0.46424E+00 0.19836E+00 0.16474E+00 0.14380E+00 0.12762E+00 0.11403E+00 0.10218E +00 0.92277E+01 0.83510E+01 0.75424E+01

YCD (SURFACE) =

0.00000E+00 0.20352E-02 0.42042E-02 0.63776E-02 0.84571E-02 0.10376E-01 0.12090E -01 0.13661E-01 0.15024E-01 0.16140E-01

XJH.

0.11677E+02 0.32624E+01 0.27297E+01 0.25095E+01 0.23858E+01 0.23039E+01 0.22435E +01 0.22180E+01 0.22112E+01 0.22049E+01

EFFICIENCY (CARBON-BALANCE)

0.00000E+00 0.18954E+00 0.29281E+00 0.37440E+00 0.44306E+00 0.50236E+00 0.55425E +00 0.60008E+00 0.64109E+00 0.67788E+00 0.68968E+00 0.69592E+00 0.70227E+00 0.70873E+00 0.71531E+00 0.72200E+00 0.72882E +00 0.73577E+00 0.74284E+00 0.75003E+00 0.75736E+00 0.76481E+00 0.77240E+00 0.78011E+00 0.78795E+00 0.79591E+00

EFFICIENCY (TEMPERATURE)

0.00000E+00 0.11074E+00 0.14304E+00 0.16233E+00 0.17485E+00 0.18292E+00 0.18775E +00 0.19010E+00 0.19058E+00 0.18953E+00 0.19407E+00 0.20031E+00 0.20666E+00 0.21312E+00 0.21970E+00 0.22640E+00 0.23322E +00 0.24016E+00 0.24723E+00 0.25443E+00 0.26175E+00 0.26921E+00 0.27679E+00 0.28451E+00 0.29234E+00 0.30031E+00

TIME= 0.59999996 KDUNT=

GAS TEMPERATURE (T)

0.10000E+01 0.10637E+01 0.10848E+01 0.10979E+01 0.11067E+01 0.11127E+01 0.11166E +01 0.11188E+01 0.11198E+01 0.11197E+01 0.11225E+01 0.11262E+01 0.11301E+01 0.11340E+01 0.11380E+01 0.11421E+01 0.11463E +01 0.11506E+01 0.11550E+01 0.11594E+01 0.11640E+01 0.11686E+01 0.11733E+01 0.11780E+01 0.11828E+01 0.11876E+01

61

SOLID TEMPERATURE (TS)

0.12635E+01 0.12134E+01 0.11791E+01 0.11578E+01 0.11419E+01 0.11289E+01 0.11179E +01 0.11088E+01 0.11011E+01 0.10940E+01

CNHM MASS FRACTION (YF)

0.10000E+01 0.79629E+00 0.68495E+00 0.59669E+00 0.52239E+00 0.45843E+00 0.40278E +00 0.35401E+00 0.31086E+00 0.27270E+00 0.25498E+00 0.24161E+00 0.22817E+00 0.21469E+00 0.20118E+00 0.18767E+00 0.17418E +00 0.16074E+00 0.14739E+00 0.13416E+00 0.12109E+00 0.10825E+00 0.95689E-01 0.83472E-01 0.71677E-01 0.60388E-01

YCD =

0.00000E+00 0.14469E-01 0.30578E-01 0.46066E-01 0.60086E-01 0.72238E-01 0.82367E -01 0.90438E-01 0.96435E-01 0.10054E+00 0.11961E+00 0.14221E+00 0.16422E+00 0.18553E+00 0.20602E+00 0.22560E+00 0.24413E +00 0.26148E+00 0.27750E+00 0.29206E+00 0.30499E+00 0.31614E+00 0.32534E+00 0.33244E+00 0.33728E+00 0.33972E+00

SURFACE HC MASS FRACTION (YFS)

0.43022E+00 0.16752E+00 0.14035E+00 0.12323E+00 0.10966E+00 0.97974E-01 0.87588E -01 0.78891E-01 0.70921E-01 0.63536E-01

YCD (SURFACE) =

0.00000E+00 0.16102E-02 0.35915E-02 0.57604E-02 0.79705E-02 0.10111E-01 0.12099E +01 0.13980E-01 0.15616E-01 0.16983E-01

XUH.

0.11677E+02 0.32865E+01 0.27628E+01 0.25486E+01 0.24291E+01 0.23501E+01 0.22915E +01 0.22746E+01 0.22690E+01 0.22633E+01

EFFICIENCY (CARBON-BALANCE)

0.00000E+00 0.20056E+00 0.30840E+00 0.39329E+00 0.46453E+00 0.52586E+00 0.57930E +00 0.62632E+00 0.66817E+00 0.70543E+00 0.71900E+00 0.72746E+00 0.73610E+00 0.74495E+00 0.75400E+00 0.76325E+00 0.77271E +00 0.78238E+00 0.79225E+00 0.80231E+00 0.81256E+00 0.82298E+00 0.83354E+00 0.84421E+00 0.85495E+00 0.86571E+00

EFFICIENCY (TEMPERATURE)

0.00000E+00 0.14356E+00 0.19111E+00 0.22064E+00 0.24056E+00 0.25404E+00 0.26275E +00 0.26777E+00 0.26994E+00 0.26975E+00 0.27609E+00 0.28455E+00 0.29320E+00 0.30204E+00 0.31109E+00 0.32034E+00 0.32980E +00 0.33947E+00 0.34934E+00 0.35940E+00 0.36965E+00 0.38007E+00 0.39063E+00 0.40130E+00 0.41204E+00 0.42280E+00 TIME= 0.80000001 KBUNT=

GAS TEMPERATURE (T)

0.10000E+01 0.10736E+01 0.11008E+01 0.11183E+01 0.11306E+01 0.11392E+01 0.11452E +01 0.11490E+01 0.11512E+01 0.11519E+01 0.11557E+01 0.11606E+01 0.11656E+01 0.11708E+01 0.11761E+01 0.11816E+01 0.11871E +01 0.11928E+01 0.11986E+01 0.12045E+01 0.12104E+01 0.12162E+01 0.12220E+01 0.12276E+01 0.12330E+01 0.12380E+01

81

14 4

SOLID TEMPERATURE (TS)

0.12842E+01 0.12637E+01 0.12263E+01 0.12021E+01 0.11836E+01 0.11683E+01 0.11551E +01 0.11442E+01 0.11347E+01 0.11258E+01

CNHM MASS FRACTION (YF)

0.10000E+01 0.78985E+00 0.67457E+00 0.58286E+00 0.50547E+00 0.43878E+00 0.38082E +00 0.33013E+00 0.28551E+00 0.24636E+00 0.22574E+00 0.20896E+00 0.19220E+00 0.17549E+00 0.15888E+00 0.14244E+00 0.12625E +00 0.11038E+00 0.94950E-01 0.80069E-01 0.65879E-01 0.52539E-01 0.40228E-01 0.29146E-01 0.19511E-01 0.11553E-01

YCD =

0.00000E+00 0.15324E-01 0.33574E-01 0.51842E-01 0.68808E-01 0.83726E-01 0.96218E -01 0.10609E+00 0.11328E+00 0.11798E+00 0.14070E+00 0.16711E+00 0.19204E+00 0.21529E+00 0.23659E+00 0.25571E+00 0.27238E +00 0.28632E+00 0.29728E+00 0.30498E+00 0.30918E+00 0.30970E+00 0.30638E+00 0.29913E+00 0.28799E+00 0.27306E+00

SURFACE HC MASS FRACTION (YFS)

0.41453E+00 0.14723E+00 0.12338E+00 0.10845E+00 0.96428E-01 0.85904E-01 0.76418E -01 0.68448E-01 0.60959E-01 0.54005E-01

YCD (SURFACE) =

0.00000E+00 0.13253E-02 0.30987E-02 0.51695E-02 0.73836E-02 0.96090E-02 0.11737E -01 0.13799E-01 0.15599E-01 0.17119E-01

NJH.

0.11677E+02 0.33030F+01 0.27876E+01 0.25795E+01 0.24646E+01 0.23889E+01 0.23328E +01 0.23243E+01 0.23206E+01 0.23162E+01

EFFICIENCY (CARBON-BALANCE)

0.00000E+00 0.20682E+00 0.31813E+00 0.40586E+00 0.47956E+00 0.54301E+00 0.59825E +00 0.64679E+00 0.68985E+00 0.72798E+00 0.74366E+00 0.75469E+00 0.76603E+00 0.77768E+00 0.78965E+00 0.80193E+00 0.81450E +00 0.82733E+00 0.84038E+00 0.85359E+00 0.86686E+00 0.88009E+00 0.89312E+00 0.90578E+00 0.91784E+00 0.92905E+00

EFFICIENCY (TEMPERATURE)

0.00000E+00 0.16600E+00 0.22721E+00 0.26670E+00 0.29434E+00 0.31389E+00 0.32731E +00 0.33593E+00 0.34078E+00 0.34242E+00 0.35100E+00 0.36203E+00 0.37337E+00 0.38503E+00 0.39700E+00 0.40928E+00 0.42185E +00 0.43468E+00 0.44773E+00 0.46093E+00 0.47421E+00 0.48744E+00 0.50047E+00 0.51313E+00 0.52519E+00 0.53639E+00

TIME= 1.00000000 KDUNT= 101

6A3 TEMPERATURE (T)

0.10000E+01 0.10805E+01 0.11129E+01 0.11345E+01 0.11501E+01 0.11616E+01 0.11698E +01 0.11755E+01 0.11792E+01 0.11811E+01 0.11861E+01 0.11922E+01 0.11986E+01 0.12052E+01 0.12120E+01 0.12189E+01 0.12259E +01 0.12330E+01 0.12401E+01 0.12470E+01 0.12536E+01 0.12597E+01 0.12651E+01 0.12696E+01 0.12730E+01 0.12757E+01

SOLID TEMPERATURE (TS)

0.12942E+01 0.13041E+01 0.12666E+01 0.12412E+01 0.12213E+01 0.12045E+01 0.11899E +01 0.11778E+01 0.11670E+01 0.11568E+01

CNHM MASS FRACTION (YF)

0.10000E+01 0.78602E+00 0.66774E+00 0.57313E+00 0.49291E+00 0.42358E+00 0.36325E +00 0.31050E+00 0.26422E+00 0.22386E+00 0.20022E+00 0.18001E+00 0.15997E+00 0.14021E+00 0.12085E+00 0.10205E+00 0.83997E -01 0.66901E-01 0.51025E-01 0.36667E-01 0.24163E-01 0.13877E-01 0.61870E-02 0.14578E-02 0.00000E+00 0.00000E+00

YCB =

0.00000E+00 0.15904E-01 0.35886E-01 0.56611E-01 0.76315E-01 0.93878E-01 0.10864E +00 0.12020E+00 0.12840E+00 0.13347E+00 0.15912E+00 0.18804E+00 0.21403E+00 0.23668E+00 0.25556E+00 0.27025E+00 0.28033E +00 0.28546E+00 0.28534E+00 0.27984E+00 0.26894E+00 0.25285E+00 0.23195E+00 0.20684E+00 0.17831E+00 0.15073E+00

SURFACE HC MASS FRACTION (YFS)

0.40722E+00 0.13348E+00 0.11127E+00 0.97579E-01 0.86488E-01 0.76671E-01 0.67731E -01 0.60191E-01 0.52984E-01 0.46285E-01

YCD (SURFACE) =

0.00000E+00 0.11340E-02 0.27210E-02 0.46606E-02 0.68086E-02 0.90267E-02 0.11191E -01 0.13321E-01 0.15180E-01 0.16751E-01

HUX

0.11677E+02 0.33143E+01 0.28063E+01 0.26039E+01 0.24936E+01 0.24215E+01 0.23683E +01 0.23678E+01 0.23657E+01 0.23641E+01

EFFICIENCY (CARBON-BALANCE)

0.00000E+00 0.21052E+00 0.32445E+00 0.41456E+00 0.49049E+00 0.55600E+00 0.61311E +00 0.66335E+00 0.70785E+00 0.74710E+00 0.76517E+00 0.77909E+00 0.79347E+00 0.80830E+00 0.82355E+00 0.83916E+00 0.85502E +00 0.87100E+00 0.88690E+00 0.90246E+00 0.91733E+00 0.93112E+00 0.94335E+00 0.95355E+00 0.96121E+00 0.96721E+00

EFFICIENCY (TEMPERATURE)

0.00000E+00 0.18145E+00 0.25441E+00 0.30314E+00 0.33837E+00 0.36422E+00 0.38283E +00 0.39570E+00 0.40395E+00 0.40818E+00 0.41940E+00 0.43332E+00 0.44770E+00 0.46253E+00 0.47778E+00 0.49339E+00 0.50925E +00 0.52523E+00 0.54113E+00 0.55669E+00 0.57156E+00 0.58535E+00 0.59758E+00 0.60778E+00 0.61546E+00 0.62146E+00

APPENDIX II

LISTING OF COMPUTER PROGRAMS FOR CHAPTER II

The computer programs for Part II (steady state) are quite similar, in structure, to those in Part I. In addition to using a three-reactions scheme as discussed in the text of Part II, only the final results are printed out (no intermediate transient).

More Detailed Instruction (for CALY1. FOR; 75):

1. Read-in data (lines 10400 - 11475):

XPHI = ϕ = fuel-air equivalence ratio

TI = T(0,0) = upstream, initial temperature (K)

XPI = p(0,0) = upstream, initial pressure (Atm.)

UIUS = U ref = upstream (outside of the bed) initial gas velocity (m/s)

XLS = reference length (m), suggested to use 0.1m (10 cm) DS = d^* = channel hydraulic diameter (m)

AOAS = $A*/A_s*$ = open-to-close area ratio

- DXX = nondimensional step size in X (space) inside the catalytic bed. It is nondimensionalized by XLS. So if XLS = 0.1 m and DX = 0.01, the dimensional step size is 0.001 m or 1mm.
- XAS = ratio of step sizes between downstream and inside-bed regions. For example, (XAS) x (DXX) is the integration step size in X - direction in the downstream space.
- DT = nondimensional time step size, recommended to be 0.2 or 0.1 for steady calculations.
- TMAX = maximum nondimensional time allowed for computation. It should be large enough to allow the steady state to be reached. For bed length less than 5 cm, TMAX = 8.001 is suggested, for bed length between 5 - 12 cm, TMAX = 12.001 is suggested.

- N = number of grid points (in X direction) in the catalytic bed. The nondimensional catalytic bed length is given by (N-1) x (DX). For example, if N = 51, DX = .01, then the nondimensional bed length is equal to 0.5. If XLS = .1m, then the dimensional bed length is .05m (5 cm).
- NAS = number of grid points (in X direction) in the downstream after-bed space. The length of the downstream distance can be calculated in a similar manner as in the above.
- IT = interval in X for printout. For example, if DXX =.01, IT = 5, the printout will be at X = 0, 0.05, 0.1, 0.15, etc. ...

CO, C1, C2, C3, C4 = C_1 , C_2 , C_3 , C_4 (See Table I) IPUTIN = 1 (always)

2. Output data (lines 37000 -38586):

The following quantities will be printed out as a function of downstream distance:

Gas and solid temperatures, Emission indices (x1000) for the original hydrocarbon, pyrolyzed hydrocarbon, total hydrocarbon and CO in the gas phase, the original hydrocarbon and CO at the wall and the combustion efficiency.

If TMAX is large enough, the above printouts are the steady-state profiles.

CALY1.FOR;75

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12-APR-1961 23:31:21.21

FAGE 1

100 200 300 ********* ****************** *** STEADY CATALYTIC COMBUSTOR *** **4**00 50Ŏ 525 537 550 575 THREE GAS-PHASE REACTION STEPS VARIABLE X STEP SIZE (BED VS. AFTER-BED SPACE) PARAMFTEK NX=300 COMMCN /ACTI/ ES0,ES1,ES2,ES3,ES4 COMMCN /ACTI/ ES0,ES1,ES2,ES3,ES4 COMMCN /TH1/XN, WH, Q15, Q25 COMMGN /TH2/ CP, GC, XPR COMMGN /UPST/ YCI, YOI COMMCN /INI/TS,YF5,YCS,YES COMMCN /B12345/FTA1,BTA2,BTA3,ETA4,BTA5 COMMCN /GAMMA/GAMA 600 800 9ŏŏ óňň 1000 1100 1200 1400 1450 1475 1500 DIMENSION YF(NX), T(NX), TS(NX), YO(NX), YFS(NX), HC(NX), U(NX) DIMENSION XKY(4), XKT(4), XKIS(4), XKC(4), YC(NX), YCS(NX) DIMENSION XJH(NX), XJD1(NX), XJD2(NX), YH2O(NX), XP(NX) DIMENSION EFF(NX), RNK(NX) DIMENSION A(NX,3), XKE(4),YE(NX),YTHC(NX),YES(NX) EQUIVALENCF (A(1,1),TS(1)), (A(1,2),YFS(1)), (A(1,3),YCS(1)) 600 1700 1 000 1850 DIMENSION & FF (NX) / RNS(NX) DIMENSION & (A(1,1),TS(1)), (A(1,2),YFS(1)), (A(1,3),YCS(1)) FORMAT (1420.10) FORMAT (1420.10) FORMAT (1420.10) FORMAT (1420.10) FORMAT (1420.10) FORMAT (1420.10) FORMAT (140, SOLID TEMPERATURE (TS)'/) 10 FORMAT (140, SOLID TEMPERATURE (TS)'/) 11 FORMAT (140, 'ISOLID TEMPERATURE (TS)'/) 12 FORMAT (140, 'ISOLID TEMPERATURE (TS)'/) 13 FORMAT (140, 'ISOLID TEMPERATURE (TS)'/) 14 FORMAT (140, 'ISOLID TEMPERATURE (TS)'/) 15 FORMAT (140, 'ISOLID TEMPERATURE (TS)'/) 15 FORMAT (140, 'ISOLID TEMPERATURE (TS)'/) 16 FORMAT (140, 'ISOLID TEMPERATURE (TS)'/) 17 FORMAT (140, 'ISOLID TEMPERATURE (TS)'/) 19 FORMAT (140, 'ISOLID TEMPERATURE (TS)'/) 20 FORMAT (140, 'INFIILL CONDITIONS (TS)'/) 21 FORMAT (140, 'INFIILL CONDITIONS (TS)'/) 22 FORMAT (140, 'INTITAL CONDITIONS (TS)'/) 23 FORMAT (140, 'INTITAL CONDITIONS (TS)'/) 24 FORMAT (140, 'INTITAL CONDITIONS (TS)'/) 25 FORMAT (140, 'INTITAL CONDITIONS (TS)'/) 26 FORMAT (140, 'INTITAL CONDITIONS (TS)'/) 37 FORM 900 3400 3500 3600 3700 3800 4000 4100 4200 4300 4400 **46**00 4700 5000 5100 5200 5300 5400 5500 5600 5700 5800 5850 5900 6000 6100 6200 6300 6600 6650 6650 6675 6687 6743 6743 6700

CALY1.FOR;75

12-APK-1981 23:31:21.21 PAGE 2 7000 ***** INPUT PARAMETERS ***** 7000 7100 7200 7300 7400 7450 7475 7500 7600 7600 ***** CHEMICAL PARAMETERS ***** CHEFICAL REACTION 0, CNHM = CNHM (GAS PHASE PYRCLYSIS) (ISOTHERMAL REACTION) CHEMICAL REACTION 1, CNHM + G2 = CO + H2C (GAS PHASE) CHEMICAL REACTION 2, CO + G2 = CO2 (GAS PHASE) CHEMICAL REACTION 3, CNHM +C2 = CO2 + H2O (SOLIL SURFACE) CHEMICAL REACTION 4, CO + O2 = CO2 (SOLID SURFACE) 7800 7900 8000 8000 8100 8200 8300 8400 8500 8600 8600 ES1 = ACTIVATION ENERGY FOP REACTION 1 (CAL/MOLE), ETC. C1 = PREEXPONENTIAL FACTOR (DIMENSIONAL) FOR REACTION 1, ETC. WHC, WCU = MOLECULAR WEIGHTS FOR CA.HM, CO G1S = HEAT OF COMBUSTICN FOR REACTION 1 (CAL/G OF CNHM) G2S = HEAT OF COMBUSTICN FOR REACTION 2 (CAL/G OF CC) GS = HEAT OF COMBUSTION FOR REACTION 3 (CAL/G OF CNHM) XN = THE NUMBER CF CARPONS IN CNHM 8600 8900 9000 9100 9200 9300 QS = Q1S + Q2S + XN + WCD / WHC9300 9400 9500 9600 9700 9800 ***** CATALYTIC BED PARAMETERS ***** XLS = BED LENGTH (M) DS = CHANNEL HYDRAULIC DIAMETER(M) SOA = CIRCUMFERENNIIAL LENTH DIVIDED BY CHANNEL AREA,=4/DS (1/M) AOAS = RATIO OF OPEN TO CLOSED AREAS CSS = HEAT CAPACITY OF SUBSTRATE (CAL/KG/K) ROS= DESITY OF SUBSTRATE (KG/ M3) 9900 0000 10100 10200 10300 READ(5,10) XPHI READ (5,10) TI READ (5,10) XFI READ (5,10) XFI READ (5,10) UIUS REAC (5,10) LS REAL (5,10) DS READ (5,10) DT READ (5,10) DT READ (5,10) DT READ (5,10) TMAX READ (5,10) NAS RFAD(5,8) IT READ(5,8) IT READ(5,8) IPUTIN SOA = 4.0/DSCN = XN**CO/WHC NPNAS=N+NAS 10400 10500 10600 10700 10000 1000 1100 11200 11300 11400 11425 11437 11450 11475 11500 11600 11650 11700 ***** OPERATING PARAMETERS ***** TI = INITIAL, UFSTREAM (I.U.) TEMPERATURE (K) UI = I.U. VELOCITY (M/S) XPI = I.U. PRESSURE (ATM.) ROI= I.U. CAS DENSITY (KG/M3) YFI = I.U. MASS FRACTION OF HC YCI = I.U. MASS FRACTION OF CC YOI = I.U. MASS FRACTION OF O2 XPHI = UPSTREAM HC/O2 EQUIVALENCE RATIO 1900 1900 2000 12100 12100 12200 12300 12400 12500 12600 12600 12000 UI = UIUS+(1.0+AOAS)/AOAS XMU = 0.000001458+SQRI(TI)/(1.0+110.4/TI) YFI=WHC*XPHI/29./((XN+XM/4.)+4.76+XPHI) XLEN1 = 1.5* SQRT (29./WHC) XLEW2=1.5* SQRT(29./WHC) XLE123=XLEW1**.667 XLE223=XLEW2**.667 12900 13000 13100 13200 13350 13350 13375 13400 13500 XLOU = XLS/UI 13600 13700 13900 13900 14000 14100 ***** GAS PROPERTY ***** CP = SPECIFIC HEAT OF GAS (CAL/KG/K) XKI = HEAT CONDUCTIVITY OF GAS AT I.U. (CAL/ M/SEC/K)

CALY1.FOR;75	12-APR-1981 23:31:21.21 PAGE 3
14200 14300 14400 14500 14500	ALPHA = 1.U. GAS THERMAL DIFFUSIVITY (M2/SEC) GC=GAS CONSTANT FOP GAS MIXTURE (JCJLE/KG/K) XLEW1 = LEWIS NUMBER FOR CNHM IN MIXTURE XLEW2 = LEWIS NUMBER FOP CO IN MIXTURE
14700 14750 14800 14900 15000 15100	XKI=.003+.000013*TI RDI = XPI/GC/TI ALPHA = XKI/RGI/CP REY = UIUS*DS*RCI/XMU
15200 15300 15350 15375	TO DESCRIBE CHEMICAL REACTION RATE EXPRESSIONS REACTION 0 (GAS PHASE ISOTHERMAL PYRCLYSIS) D(CNHM)/DT \$\$\$ (CNHM)/EXP(E0/RT)
15400	REACTION 1 (GAS PHASE) D(CNHM)/DI \$\$\$ XF**.3*I*SORI(YHC)*YO2/EXP(F1/RT)
15000 15700 15800	REACTION 2 (GAS FHASE) D(CO)/D1 \$ P**BIA1*CO**PIA2*02**BIA3*I**BIA4*H20**PIA5/EXP(E2/RI)
16000 16100	REACTION 3 (SOLIE SURFACE) D(CNHM)/DT \$\$\$ YHCS/EXF(E3/RT)
16200 16300 16400	REACTION 4 (SOLID SURFACE) D(CC)/DT \$\$\$ YCGS/EXP(E4/RT)
16500 16600 16700	TO EVALUATE KINETIC RATE CONSTANTS.
16750 16800 16900 17000 17100 17150 17200 17300 17300 17400	B0=C0*XLOU*(WHC/ROI/YFI)**(1.=GAMA) B1SS = C1*SQRT(WHC)/WC2/SGRT(1000.) B1S = B1SS*XPI**.3*RCI*SQRT(RCI)*TI*SQRT(YFI)*Y01 B1 = XLOU*B1S/RCI/YFI H235=BTA2+HTA3+BTA5 C2M=C2*1000.**(1.=B235) B2SS=C2M*wCC**(1.=BTA2)/(WO2**ETA3*wH2O**BTA5) B2S=E2SS*XPI**ETA1*ROI**B235*T1**HTA4*YFI**BTA2*Y0I**BTA3 H2 = D2\$*XLOU/RCI/YFI
17500 17900 18000 18100 18250 18250 18275 18287 18287 18300 18500 18500 18600 18600 18600	B3=C3*DS/ALPHA/wHC/3.66 B4=C4*DS/ALPHA/wCO/3.66 Q1 = Q1S/CP/T1 Q2 = U2S/CP/TI TAE=YF1*U+1.0 TABDM=TAB*TI E0= ES0/T1/1.987 E1= ES1/T1/1.987 E2= ES2/11/1.987 E3= ES3/T1/1.987 E4= ES4/T1/1.987 AD2UL = (ALPHA/ES/DS) /(U1/XLS)
19000 19100 19200 19300 19400 19500 19500	WRITE (6,21) N,NAS WRITE(6,20) TMAX WRITE(6,15) YCI, YFI, XPHI WRITE(6,11) UIUS WRITE(6,72) UI WRITE (6,72) UI WRITE (6,17) G1, G2, DXX, DT WRITE (6,402) TI.XPR.REY, AD2UL, ADAS
19650 19675 19700 19800 19800	WRITE(6,409) TAB WRITE(6,410) TABCM WRITE(6,403)ES0,ES1,ES2,ES3,ES4 WRITE(6,40,E1,E2,E3,E4 WRITE(6,35)C0,C1,C2,C3,C4 WRITE(6,411) BTA1,BTA2,BTA3,BTA4,BTA5
20100 20200 20300 20400 20500	WRITE (6,49) ROI, XLEWI, XLEW2, XLS, DS WRITE (6,49) ROI, XKI, CF, ALPHA WRITE (6,401) XFI WRITE (6,43) XLCU WRITE (6,44) BIS, B2S WRITE(6,45)B0,B1,B2,B3,P4
1200	********
21500 21600 21650	******* UPSTREAM CONDITIONS YF(1)=1. YF(1)=0.

CALY1.FOR;75 12-APR-1981 23:31:21.21 PAGE 4 21700 21800 22000 222100 22250 22250 22250 22250 222400 23100 YC(1) = 0.0 YC(1)=1. T(1)=1. RO(1)=1. XP(1) = 1.0 U(1)=1. YTHC(1) = YF(1)+YE(1) 23100 23200 23300 ******** 200 23550 23600 INTEGRATION OF GUASI-STEADY GAS PHASE EQUATIONS FOR YF (FUEL MASS FRACTION) AND T (TEMPERATURE) IN DOWNSTREAM DIRECTION USING FOURTH-GRDER RUNGE-KUTTA SCHEME I IS THE INDEX FOR DOWNSTREAM DISTANCF X 23700 23800 23900 24000 24100 24200 24300 DO 201 I=1,NPNAS 24350 24362 24374 24375 24400 4500 24600 24700 24800 EVALUATE XJH AND XJD AS A FUNCTION OF X ******************** NDNDIMENSIUMAL HEAT CCNDUCTIVITY XK = (.003+ 0.000013+ TI +I(I)) /XK1 1.32 CONTINUE M=1 DO 120 L=1,4 IF (M.NE.1) GO TC 102 YFF= YF(I) YCC =YC(I) TI = T(I) M=M+1 GO TO 119 102 CONTINUE IF (M.NE.2) GC TO 103 YFF=YF(I)+DX*XKY(1)/2. YCC =YC(I) +DX*XKC(1)/2. TT =T(I)+DX*XKT(1)/2. 1.32 CONTINUE

ALY1.FOR;75		12-APR-1981 23:31:21.21	PAGE 5
200	M=M+1		
400 103	CONTINUE		
500	IF(M.NE.3) GO TC 10	4	
650	YEL= YE(I)+DX+XKE(2)/2	
800	YCC =YC(1) +DX+XKC(TT=1(1)+DX+XKT(2) /	2)/2.	
900	M=M+1		
100 104	CONTINUE		
9200 9250	YEF = YF(I) + DX*XK YEF= YE(T) +DX*XKE(Y(3)	
9300	¥ČC =¥Č(I) +DX*XKČ(3 j	
500 119	CONTINUE		
9600	YH20 = MASS FRACTIO	N CE H2C D*VET* (VTHC(1)=VTHC(T))/#HC	
	IF (YFF.LT.0.0) YFF		
875	IF (YEE.LT.0.0) YEE=	0.0	
9900 0000	<pre>W1 = B1* XP(I)**.3* 1 /FXP(F1/TT)</pre>	RO(I) *SQRT(RO(I)) *T(I) *SURT(YEE) *YO(I)
025	IF(YCC, LT. 0.0) YCC=	0.0	
0075	R0235=R0(1)**B235		
0100	W2=B2*R0235*SSYC*YC XKY(L) = -XJD1(I) *	C**BTA2*YH2O(I)**B1A5*XP(I)**B1 (YFF= YFS(I))/DNASS ==60/DMASS	A1/EXP(L2/IT)
0250	$XKE(\overline{L}) = -XJDI(\overline{L}) + (YE)$	É-YES(I))/DMÁSS+(WO-WI)/DMÁSS	
0400	XKT(L) =-XJH(1)*(TT	-TS(1))/CMASS +G1*YFI*W1/DMASS+	Q2+YFI+w2/DMASS
0500 120 0600	CONTINUE YF(I+1) =YF(I) +DX*	(XKY (1)/6. +XKY (2)/3. +XKY (3)/3.+XKY (4)/6.
0700	IF (YF(1+1), LT, 0, 0)	YF(1+1) = 0.0 KF(1)/6 + YFF(2)/2 + YFF(3)/2 + YFF(3	
0775	IF(YE(1+1).L1.0.0)	YE(1+1)=0.0	
0900	IC(1+1) = IC(1) + DX = IF (IC(1+1), LT.0.0)	(XKC (1)/6. + XKC (2)/3. + XKC (3)) YC(I+1) = 0.0)/3.+XKC (4)/6.)
1000	T(I+1) = T(I) + DX *	(XKI (1)/6. +XKI (2)/3. +XKT (3)/3.+XK1 (4)/6.)
1125	ŶTHĊ(Ì+1)=ŸF(Ì+1)+Ÿ	E(I+1)	
1200	YD(]+1) =1.0-XPhl*(1.0-YF(I+1))+ADD	
1300	XP(I+1)=XP(I) RD(I+1)=XP(I+1) /T(T+1)	
1500	U(1+1) = DMASS/1(1+1		
1600 201	CONTINUE	IC(I)+G2/0	
1700	END OF ONE BUNGE-KU	TTA INTEGRATION CYCLE FOR QUAST	-STFADY
1900 2000	GAS PHASE EQUATIONS	*****	
2100			
2300	*******	******	
2400	INTEGRATION OF UNST	EADY SOLID HEAT TRANSFER EQUATI	ON • OPDER
2600	RUNGE-KUTTA SCHEME	ENVIORE Y IN TIME DOING FOORTH	UNDER
2800			
2900 3000	DD 301 J =1.N		
3100	NJ=1		
3300	IF (MJ.NE.1) GU IO	302	
3400 3500	TSS =TS(J) MJ=MJ+1		
3600	GO TO 319 CONTINUE	,	
300	IF(MJ.NE.2) GD TO	303	
000	-155 = 15(J) + D1*XK -MJ=MJ+1	18(1)/2.	
	GO TO 319 Continue		
300	IF(AJ.NE.3) GO TU	304	
500	MJ=MJ+1) / 4 •	
600 700 304	GD 10 319 Continue		
100	TSS = TS(J) + DT+ X	KTS(3)	

CALY1.FOR;75 12-APR-1991 23:31:21.21 PAGE 6 YFSES=YFS(J)+YES(J) XKTS(LJ)=RNK(J)*(T(J)-ISS) +B3*YFI*Q*RO(J)*YFSES/EXP(E3/ISS) 1 +B4*YFI*Q2*RO(J)*YCS(J)/FXP(E4/ISS) 320 CONTINUE TS(J) =IS(J) +DI*(XKTS(1)/6.+XKTS(2)/3.+XKIS(3)/3.+XKTS(4)/6.) YFS(J) = YF(J) /(1.+E3*RO(J)/XLE123/RNK(J)/EXP(E3/IS(J))) YES(J)=YE(J)/(1.+E3*RO(J)/XLE123/RNK(J)/EXP(E3/IS(J))) YCS(J) = YC(J) /(1.+E4*RO(J)/XLE123/RNK(J)/EXP(E4/IS(J))) 301 CONTINUE 35100 35200 35300 35400 35450 355500 35700 35800 36200 36300 ******** LUUP FUR MARCHING FORWARD IN TIME KOUNT =KOUNT+1 KOU =KOUNT-1 XKOUNT =KOUNT-1 TIME=(XKOUNT-1.)*DT IF (TIME.GT. TMAX) GO TO 999 TIME*TMAX-DT IF (TIME.LT.TIMER) GC TO 200 WRITE(6,22) TIME ,KOUNT WRITE(6,24) (T (I),I=1,NPNAS,IT) WRITE(6,24) (T (I),I=1,N,IT) WRITE(6,24) (YF(I),I=1,NPNAS,IT) WRITE(6,24) (YF(I),I=1,NPNAS,IT) WRITE(6,24) (YF(I),I=1,NPNAS,IT) WRITE(6,24) (YTHC(I),I=1,NPNAS,IT) WRITE(6,24) (YC(I),I=1,NPNAS,IT) WRITE(6,24) (YC(I),I=1,NPNAS,IT) WRITE(6,24) (YCS(I),I=1,N,IT) WRITE(6,24) (YCS(I),I=1,N,IT) WRITE(6,24) (YCS(I),I=1,N,IT) WRITE(6,24) (YCS(I),I=1,N,IT) WRITE(6,24) (EFF(I),I=1,NPNAS,IT) WRITE(6,24) (FF(I),I=1,NPNAS,IT) WRITE(6,24) (FF(I),I=1,N,IT) WRITE(6,24) (FF(I),I=1,N,IT) WRITE(6,24) (FF(I),I=1,N,IT) WRITE(6,24) (FF(I),I=1,N,IT) WRITE(6,24) (FF(I),I=1,NPNAS,IT) WRITE(1,*) A CLOSE (UNIT=11) GD TC 200 END CF WHOLE INTEGRATICN LCOF LOOP FOR MARCHING FORWARD IN TIME 6950 3730Ò 37500 37675 37700 780ŏ **\$0**00 ŏ \$200 ð 3 Ó Ö ¥543 38593 38599 38624 38650 38700 38800 39000 39100 39200 39300 39400 39500 39500 601 CONTINUE WRITE (6,404) WRITE(6,405) 999 CONTINUE STOF END

BLOCK FOR; 28

12-AFR-1981 23:33:05.76

100 BLOCK DATA
200
300 DIMENSION TS(300),YFS(300),YCS(300),YFS(300)
400 COMMCN /AC11/ESO,ES1,ES2,ES3,ES4
600 COMMCN /AC11/ESO,ES1,ES2,ES3,ES4
700 COMMCN /IH1/XN, XM, 01S, 02S
600 COMMCN /IH1/XN, XM, 01S, 02S
600 COMMCN /UP5T/ YCI, YGI
1100 COMMCN /IN1/TS,YFS,YCS,YES
1150 COMMCN /H12345/BTA1,ETA2,ETA3,ETA4,ETA5
1175 COMMCN /GAMMA/GAMA
1200
1400 DATA ESO,ES1,ES2,ES3,ES4/34000.,24000.,40000.,10000.,17800./
1500 DATA WHC,WC0,WC2,WH20/138.,28.,32.,18./
1600 DATA XN,XM,01S,G2S/10.,18.,5357000.,2415000./
1700 DATA CP.GC,XPR/285,2.4312L-3,0.7/
1800 DATA ATA CP.GC,XPR/285,2.4312L-3,0.7/
1900 DATA ATA GAMA/U.5/
1900 DATA GAMA/U.5/
2100 END

FAGE 1

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Example calculation:

\$ RUN CALY1.EXE;61 .165 ŤI. 1000. X PI з. UIUS 15. XLS . 1 .001382 DS Read-in 13.29 AOAS data .01 1. DXX, XAS .1 DT 8.001 TMAX 51 NAS 100 5 IT 1.6E6 1.8E5 .54E14 1.5E4 1.E5 co, c1, c2, C3, c4 1 IPUTIN ND. DF GRID PDINT = 51100

TMAX = 8.00100040

YD(UPST,INI) = 0.230000 YF (UPST,INI) = 0.0113489 EQUI RATIO = 0.165 000007

APPRDACH VEL = 15.0000

CHANNEL VEL (I.U.) = 16.12867

.

01= 18.7964916 Q2= 8.4736843 DX= 0.0100000 DT= 0. 1000000

TI, PR, REY, AD2UL, ADAS = 1000.00000 0.70000 529.01978 0.17199 13 .29000

ADIABATIC FLAME TEMP = 1.408440

ADIAB FLAME TEMP(K) = 1408.44019

ES+S= 0.34000000E+05 0.24000000E+05 0.4000000E+05 0.100000 00E+05 0.17800000E+05

E.S = 17.111223 12.078510 20.130850 5.032712 8.958228

C+S = 0.160000000E+07 0.180000000E+06 0.5399999886E+14 0.1500 000000E+05 0.100000000E+06

BETA 1-5 = 0.0000 1.0000 0.2500 0.0000 0.5000

6C, XLEW1, XLEW2, XLS, DS = 0.002831 0.687624 1.526551 0.100000 0 .001382

FDI, XKI, CP, ALPHA = 1.059621 0.016000 285.000000 0.52982E-04

XPI = 3.00000

L/U = 0.0062001

B1S, B2S, = 0.77647750E+05 0.26173910E+09

B0-4 = 0.10626966E+07 0.40033785E+05 0.13494798E+09 0.774666638E+03 0.25453324E+0 5 1

TIME= 8.00000000 KOUNT= 81

GAS TEMPERATURE (T)

0.10000E+01 0.10933E+01 0.11328E+01 0.11648E+01 0.11927E+01 0.12180E+01 0.12416E +01 0.12640E+01 0.12855E+01 0.13063E+01 0.13262E+01 0.13387E+01 0.13506E+01 0.13629E+01 0.13739E+01 0.13806E+01 0.13853E +01 0.13891E+01 0.13923E+01 0.13948E+01 0.13969E+01 0.13985E+01 0.13998E+01 0.14008E+01 0.14016E+01 0.14022E+01 0.14027E +01 0.14031E+01 0.14034E+01 0.14036E+01 0.14038E+01

SOLID TEMPERATURE (TS)

0.11286E+01 0.13193E+01 0.13325E+01 0.13421E+01 0.13502E+01 0.13576E+01 0.13646E +01 0.13713E+01 0.13779E+01 0.13842E+01 0.13901E+01

DRIGINAL CNHM (YF)

0.10000E+01 0.76186E+00 0.65814E+00 0.57097E+00 0.49212E+00 0.41866E+00 0.34867E +00 0.28172E+00 0.21825E+00 0.15913E+00 0.10576E+00 0.65733E-01 0.33518E-01 0.10262E-01 0.16875E-03 0.00000E+00 0.00000E +00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E +00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00

FYRELYZED CNHM (YE)

0.00000E+00 0.37022E-02 0.91331E-02 0.15642E-01 0.22801E-01 0.30127E-01 0.37036E -01 0.42908E-01 0.47042E-01 0.48627E-01 0.46778E-01 0.44312E-01 0.36474E-01 0.22910E-01 0.59886E-02 0.00000E+00 0.00000E +00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E +00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00

TOTAL CNHM (YTHC)

0.10000E+01 0.76556E+00 0.66727E+00 0.58662E+00 0.51493E+00 0.44878E+00 0.38571E +00 0.32463E+00 0.26529E+00 0.20776E+00 0.15254E+00 0.11004E+00 0.69992E-01 0.33172E-01 0.61574E-02 0.00000E+00 0.00000E +00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E +00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00

YC**D** =

0.00000E+00 0.16567E-02 0.65324E-02 0.15098E-01 0.27740E-01 0.44585E-01 0.65277E +01 0.89093E-01 0.11487E+00 0.14085E+00 0.16454E+00 0.21374E+00 0.26066E+00 0.28957E+00 0.28939E+00 0.24645E+00 0.19836E +00 0.15831E+00 0.12542E+00 0.98769E-01 0.77393E-01 0.60398E-01 0.46983E-01 0.36453E-01 0.28226E-01 0.21821E-01 0.16848E -01 0.12996E-01 0.10017E-01 0.77166E-02 0.59417E-02

SURFACE HC MASS FRACTION (YFS)

0.59588E+00 0.55527E-01 0.41276E-01 0.33799E-01 0.28303E-01 0.23840E-01 0.20075E -01 0.16396E-01 0.12835E-01 0.94545E-02 0.63485E-02

YCD (SURFACE) =

0.00000E+00 0.12246E-03 0.40424E-03 0.86451E-03 0.15176E-02 0.23790E-02 0.34733E -02 0.47282E-02 0.60816E-02 0.74420E-02 0.86843E-02

XJH.

0.42719E+02 0.39692E+01 0.33866E+01 0.31814E+01 0.30823E+01 0.30479E+01 0.30827E +01 0.31189E+01 0.31556E+01 0.31922E+01 0.32283E+01

EFFICIENCY (CARBON-BALANCE)

0.00000E+00 0.23405E+00 0.33119E+00 0.40983E+00 0.47854E+00 0.54072E+00 0.59893E +00 0.65439E+00 0.70766E+00 0.75908E+00 0.80872E+00 0.83963E+00 0.86864E+00 0.89865E+00 0.92571E+00 0.94197E+00 0.95330E +00 0.96273E+00 0.97047E+00 0.97674E+00 0.98178E+00 0.98578E+00 0.98894E+00 0.99142E+00 0.99335E+00 0.99486E+00 0.99603E +00 0.99694E+00 0.99764E+00 0.99818E+00 0.99660E+00

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16. Abstract	······································			······································					
			wood as analyses						
A quasisteady-gas+phase and	i inermally-inin-s	substrate model 15	used to analyze i	the transient					
behavior of catalytic monolith o	combustors in fuel	-lean operation.	The combustor rea	sponse delay					
is due to the substrate thermal	inertia. Fast re	sponse is found to	be favored by th	hin substrate,					
about astalutio had larath high	is due to the substrate thermal increta, rust responde is found to be involved by this substrate,								
snort catalytic ded length, high	short catalytic bed length, high combustor inlet and final temperatures and, in most cases,								
small gas channel diameters. Th	small gas channel diameters. The calculated gas and substrate temperature time history at								
different axial positions provid	les an understandi	ing of how the cata	lytic combustor :	responds					
to an unstream condition change	The computed re	sults also suggest	that the was re	sidence times					
to an upstream condition change. The computed results also suggest that the gas residence times									
in the catalytic bed and in the	after-bed space a	ire correlatable wi	th the nondimens	ional compustor					
response time. The model can al	lso perform steady	-state combustion	calculations; and	d the computed					
steady-state emission character:	istics show good	greement with avai	lable experiment	al data in					
	From combined	teady and unstandy	considerations	it is possible					
the range of parameters covered.	. From compined (sceauy anu unstead)	considerations,	LE LO POBDIDIC					
to choose a catalytic combustor	design for automo	otive gas turbine e	ngine which has	reasonably					
fast response (< 1 second) and	can satisfy the	mission goals in a	in acceptable tot	al combustor					
length.	length								
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