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Development of a CFD Code for Casting Simulation

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Table of Contents

	Page
List of Tables	ii
List of Figures	iii
1.0 Introduction	1
2.0 Technology Search	3
2.1 Literature Search	5
2.2 Code Search	6
2.3 Consortium Activities	8
3.0. Evaluation of Codes	13
4.0. Benchmarking of ProCAST Code	15
4.1 Backward-Facing Step	17
4.2 Duct Flow	22
4.3 Volume Filling, Turbulence, and Waves	25
5.0 Conclusions	27
6.0 Recommendations	28
Bibliography	49

List of Tables

	Page
Table I. Survey of Casting Simulation Codes.....	29
Table II. Attendees List.....	32
Table III. Predicted Detachment and Re-Attachment Locations From Backward Facing Step Solutions.....	33

List of Figures

	Page
Figure 1. Backward-Facing Step 2-D Flowfield Geometry	34
Figure 2. Experimental Results from Reference 149	35
Figure 3. Predicted Results from Reference 149	35
Figure 4. Finite Element Meshes for Backward-Facing Step	36
Figure 5. Inlet Average Velocities for Backward-Facing Step	37
Figure 6. Velocity Vectors for the Medium Mesh, $Re = 100$ Case	38
Figure 7. Velocity Contours for the Fine Mesh, $Re = 1,200$ Case	39
Figure 8. Predicted Results versus Reference 149 Experimental Results	40
Figure 9. Predicted Results versus Reference 149 Predicted Results	40
Figure 10. Duct Flow Finite Element Mesh from PATRAN	41
Figure 11. Packed Mesh for Duct Flow	42
Figure 12. Comparison of Current Mesh and Reference 150 Mesh	43
Figure 13. Inlet Velocities for Uniform Mesh Duct Flow	44
Figure 14. Inlet Velocity Profiles for Packed Mesh Duct Flow	45
Figure 15. Velocity Magnitude Contours for Uniform Mesh Duct Flow	46
Figure 16. Velocity Magnitude Contours for Packed Mesh Duct Flow	47
Figure 17. Radial (y-direction) Velocity Contours for Uniform Mesh Duct Flow	48
Figure 18. Streamwise Velocity Profiles for Duct Flow	49
Figure 19. Velocity Profiles Across the Span (z-direction for Duct Flow)	50
Figure 20. Mold Filling Flowfield Demonstration	51
Figure 20. Mold Filling Flowfield Demonstration, continued	52
Figure 21. Laminar Solution of Turbulence Case	53

1.0 Introduction

Because of high rejection rates for large structural castings (e.g. the Space Shuttle Main Engine Alternate Turbopump Design Program), a reliable casting simulation computer code is very desirable. This code would reduce both the development time and life cycle costs by allowing accurate modeling of the entire casting process. While this code could be used for other types of castings, the most significant reductions of time and cost would probably be realized in complex investment castings, where any reduction in the number of development castings would be of significant benefit.

The casting process is conveniently divided into three distinct phases:

- 1) mold filling, where the melt is poured or forced into the mold cavity,
- 2) solidification, where the melt undergoes a phase change to the solid state, and
- 3) cool down, where the solidified part continues to cool to ambient conditions.

While these phases may appear to be separate and distinct, temporal overlaps do exist between phases (e.g. local solidification occurring during mold filling), and some phenomenological events are affected by others (e.g. residual stresses depend on solidification and cooling rates). Therefore, a reliable code must accurately model all three phases and the interactions between each. While many codes have been developed (to various stages of complexity) to model the solidification and cool down phases, only a few codes have been developed to model mold filling.

The current task involves developing a computational fluid dynamics (CFD) code to accurately model the mold filling phase. This task is being accomplished using a systematic approach, which includes a technology search, an evaluation of existing codes and a code development effort. The technology search includes a literature search, a code search and participation with casting industry personnel and officials involved in casting consortium start-ups. The literature search, while not exhaustive, is comprehensive and includes both technical and informative material covering all phases of casting processes and modeling. While much of the literature described available casting simulation codes, additional literature and material was obtained from the code developers and code users as a part of the code search. From this material and inputs from industry personnel, an evaluation of these codes was made to determine their suitability for continued development into a reliable, accurate and comprehensive casting simulation code. As a result of this evaluation, a decision was made that development of a new CFD code was not cost effective or necessary. The approach selected, and supported by the casting industry, was to support further development of existing codes. Also with industry concurrence, ProCAST (developed and marketed by UES) was selected for further development by means of providing an independent evaluation of the code's casting simulation capabilities. The first step of

this process was to evaluate the mold filling analysis capabilities. This should be extended in subsequent efforts to include evaluations of the solidification and cool down analysis capabilities.

The ProCAST code was used to model several different transient and steady-state fluid flow cases, some of which have been used previously to benchmark other CFD codes. The first case involves air flowing through a channel with a backward-facing step. The second case involves water flowing through a square duct turning a 90° bend. Others involved filling transients, turbulence, and wave propagation. The results of the ProCAST analysis of each case are compared to both test data and previous analytical results. Most of the results agree well with test data and predictions, where available. Most of the discrepancies are easily attributable to limitations in the models chosen. While much more complex models could be used, requiring much more set-up time, CPU time, computer storage requirements and post-processing time, the fluid models are probably already more complex than would normally be used for most casting simulation analyses. Moreover, the intent of the evaluation process is not to rigorously exercise the code using supercomputer capabilities, but to determine the code's capabilities on smaller computer systems (such as the SGI Personal Iris system used here) using reasonably sized models normally used in casting simulations, where the flowfield is only a portion of the overall simulation. Furthermore, most casting companies that will be using this code do not have access to a supercomputer, and even if they did, it may not be cost effective.

Recommendations are included to identify future efforts to be accomplished before the ProCAST code can be used as a reliable casting simulation code to support casting and quality issues.

2.0 Technology Search

While the technology search includes primarily the literature search, the code search and the consortium activities, other meetings and discussions with casting industry personnel contributed to the overall information gathering process. The literature search, code search and consortium activities are discussed in the following subsections. A brief description of a few of the other meetings and discussions is included here.

A casting simulation meeting, which served as a kick-off for this project, was attended at MSFC. Personnel from Howmet Corporation and Pratt & Whitney made presentations to MSFC and ERCI attendees.

- Mr. Jan Lane, Technical Manager, Howmet, Hampton, VA, gave an informative briefing on the investment casting process, identifying many of the problems encountered and how they relate to lack of understanding the mold filling and solidification processes. Ninety (90) percent of today's investment casting problems are related to hot spots in the mold (shell) created during mold filling. Fluid flow simulation of this process could identify and alleviate many of the problems.
- Dr. John. S. Tu, Staff Engineer, Howmet, Whitehall, MI, presented examples of solidification simulation recently performed using TOPAZ. He discussed many of the problems encountered during solidification and the need for better modeling/simulation capabilities.
- Mr. Rick Montero, Pratt & Whitney, presented a Structural Casting Process Modeling Technology Development Program, identifying a very comprehensive effort to improve the ability to model castings.

Contact was made via telephone with Tom Glasgow, Chief, Processing Science and Technology Branch, NASA/Lewis Research Center (LeRC). Discussions revealed that they are/have:

- formed a multi-disciplinary Computational Materials Laboratory
- analysis/performance of solidification processes, CVD, etc.
- evaluated several codes - FIDAP, FLUENT, NEKTON,....
- funding FIDAP improvements through an SBIR
- funding create.x to improve phase change chemistry in FLUENT

Subsequent to this discussion, MSFC personnel (Dr. Paul K. McConnaughey/ED32 and Dr. Biliyar Bhat/EH23) visited LeRC to ensure that the current effort is synergistic with their efforts and is not redundant or represent excessive overlap of technical assignments.

Most of the literature and code search efforts were accomplished near the beginning of the effort with only limited updates afterward. Therefore, these efforts are

not as current or complete as they could be. Any extended effort should include an update as soon as possible.

2.1 Literature Search

The literature search was a multi-purpose effort to identify what has been and is being done in the area of casting simulation, especially investment castings. Of the three casting phases (mold filling, solidification and cooldown), emphasis is placed on the first two, especially where the melt is still liquid and fluid flow simulation is applicable. Additionally, references are located which have general interest to the fluid flow (mold filling) simulation process and the solidification process, even for some time after the melt has solidified.

In the process of acquiring references both directly and indirectly related to the simulation process, much general information on casting processes and casting technology was also found. The references obtained are divided into three categories in the attached Bibliography:

- Mold Filling (53 references, No. 1 through 53)
- Solidification (31 references, No. 54 through 84)
- General Casting Technology (64 references, No. 85 through 148)

While many references are of a technical nature, many are semi-technical and some are non-technical, which were included in the search since the intent was to learn as much as possible about casting processes, terminology, materials, innovations, industry perceptions, etc. that could prove useful during the code development process. Most of these references were used as a tool for establishing the current state-of-the-art of casting simulation; therefore, a large portion of a synopsis of the references would read like a tutorial of casting processes and problems, and is not included here.

2.2 Code Search

The intent of the code search was to identify all available CFD codes with mold filling capabilities as well as the best solidification codes available. The mold filling codes will be evaluated to determine the need for a new code or further development of one of the existing codes. Knowledge of the solidification codes will provide a basis for future development in this area, especially if it were determined that a new CFD mold filling code needed to be developed which would need to interface with one of the solidification codes.

The search for casting codes was aided by the literature search, the consortium activities, and other communications. The casting codes identified are very briefly described in Table I. While many of the codes are solidification only with no mold filling capabilities, they are included here for previously described interests. Emphasis is placed on the mold filling codes, which span a broad range of complexity, from very simple (and, consequently, of very limited use) to very complex. The details of the more complex codes (user techniques, mathematics, physics, etc.) are usually proprietary, with only limited marketing information available. However, general descriptions of the codes' methodologies are not proprietary, and much additional information has been gained through telephone conversations, marketing, and other literature and personal visits, as described below. The following is a brief synopsis of information obtained on the mold filling codes:

1. ProCAST, developed and marketed by UES, Inc. (reference 3), is touted by many people in the casting industry as the only finite element code that simulates mold filling. Actually, there are other FE codes, such as NEKTON, but ProCAST is the only one to have been developed specifically for mold filling that solves the full, unsteady Navier Stokes equations, and includes a $k-\epsilon$ turbulence model.

Dr. Mark Samonds, who directs the ProCAST development group at UES, visited MSFC on December 5, 1991, and discussed the technical details of the code. While most of the details of the code are presented in reference 2 (previously provided by Dr. Samonds), the presentation and discussions/questions were very informative. Also discussed were other codes (Samonds says Magmasoft is his biggest competition) and pre-processors (ProCAST uses PreCAST which interfaces with PATRAN, IDEAS or ANVIL). Dr. Samonds also provided a list of current users of the latest version of ProCAST containing the mold filling simulation capability. I have contacted a few of these, but they have so far used the fluid dynamics version of ProCAST on very limited applications. Several plan to use the code more extensively and a follow-up survey should be conducted at a later date.

2. Magmasoft, developed and marketed by Magma (Germany), is a finite difference code about which little is known. Since this code is said to be the strongest

competition for ProCAST, more information about the code and its users should be obtained.

3. Simulor, developed and marketed by Aluminium Pechiney of Voreppe, France, is a finite difference mold filling code which solves the full, unsteady Navier Stokes equations. References 4, 5, and 39 provide some limited information about the code along with some examples of applications of both mold filling and solidification. The code contains a few numerical options (numerical simplifications/approximations) such as "free surface smoothing" and the "false transient approach" to decrease execution times. No American users of the code have yet been located.
4. RaPiDcast, developed by Metalworking Technology, Inc. (MTI), under a U. S. Navy contract. is to be marketed by a third party. The code is finite difference, based on SOLA-VOF, and is an extension/variation of the Ph.D. dissertation work that Dr. C. Wang performed at the University of Pittsburgh under the direction of Prof. Robert Stoehr. The solution technique is time-accurate explicit, although an implicit version is currently being written. The R, P, and D in RaPiDcast represent an acronym for "Rational Process Design," which is the philosophy of MTI, a non-profit subsidiary of the University of Pittsburgh. MTI continues to improve both the CFD and the solidification segments of the code. (Information obtained from Mike Tims and Dr. Anand Paul of MTI). See Reference 38.
5. FLOW3D, marketed by Flow Science, Inc., is (according to the users I have talked to) a very versatile finite difference code which gives good numerical results, but is not user friendly and requires long execution times. A copy of a marketing brochure was obtained from Dr. John Tu of Howmet. See References 8 and 9.
6. FLOCAST, developed at the University of Pittsburgh under the direction of Prof. Robert Stoehr, uses the finite difference SOLA-VOF method, as does RaPiDcast. Not much is known about FLOCAST at this time. Additional information is needed.
7. NEKTON, developed by Nektonics, Inc., marketed by create.x, uses the FE spectral element method. The marketing brochure (reference 40) indicates that the code will handle Boussinesq natural convection, creeping flows, and other phenomena, varying boundary conditions, and special applications. Casting simulation is only one of the many applications advertised. Therefore, it could suffer from too much generality (as FLOW3D), resulting in less user friendliness and long run times.

2.3 Consortium Activities

Discussion with industry personnel identified a number of consortium activities; some are well established and some are in the initial (startup) phase. It was decided that participation in these activities would be of benefit in establishing industry direction in casting simulation efforts. Of primary interest were the Sandia FASTCAST and the NIST/NCAT consortiums, for which large portions of the consortiums activities are directed toward modeling/simulation efforts. These activities are described below.

Sandia FASTCAST Consortium

The Sandia National Laboratories Investment Casting Workshop, held November 5 - 6, 1991, was filled with informative presentations, discussions, and tours. The purpose of the workshop was to inform the casting industry of the work that Sandia has been doing, their future plans, and their idea of organizing a consortium to compliment and transfer this technology. The workshop was well organized, well planned, and well attended (I estimate approximately 50 industry attendees and approximately 30 Sandia participants). Other than two presentations on investment casting perspectives by industry officials, the remainder of the workshop was conducted by Sandia personnel with Mr. Frank Zanner as the organizer and moderator. All the Sandia presentations highlighted the organized effort to develop the FASTCAST Process which includes the following technology areas:

- Rapid prototyping
- Solid modeling
- Rules for casting
- Systems integration
- Numerical simulation

[While our primary interest is in the area of numerical simulation, we must also be familiar with and interface with the other disciplines.]

Rapid prototyping involves techniques for quickly manufacturing an investment pattern without the need for a mold. Several methods and equipment are available for this process, the most promising being:

- Stereolithography, which is a three-dimensional process which solidifies liquid photo-curable polymer into a programmed shape.
- Selective Laser Sintering, where a thin layer of wax/refractory powder is laser-sintered into a programmed shape.
- Fused Deposition Modeling, where a thermoplastic material is extruded onto the model in thin layers which are then bonded by thermal heating.

All three processes can access CAD data files. Other methods are also being considered by Sandia for further study.

Solid modeling of the part to be cast on a CAD system is necessary to enable automated mesh/grid generation for numerical simulation and for rapid prototyping. Most users of these systems are not satisfied with their current capabilities and ease of use (user friendliness).

The "rules for casting" is a part of the Casting Toolkit, which also includes experimental data, design rules, and design history. Sandia has run a series of casting experiments to determine the fluidity of the melt when filling thin wall sections. While the results of these experiments are to date incomplete, they should eventually lead to useful design/casting rules for the Casting Toolbox. The consortium would identify and fund more technology efforts such as this.

The system integration technology "simply" ties all the various modeling, design, analysis, Casting Toolkit, rapid prototyping, etc. together in an orderly process. This is not a simple task.

The numerical simulation is the technology of most interest since this includes the thermal, structural, and most especially, the fluid flow modeling of the mold filling process. Sandia is now in the process of developing a system of codes and methodology to accomplish complete modeling of the mold filling, solidification, and cooldown of the part and the mold.

The currently proposed fluid flow code is NACHOS II unless a better code is found. No details of the code's methodology or capabilities were given in the presentation. However, it was stated that the least amount of time was spent on this portion (fluid flow) part of the analyses. The fluid flow modeling is to be started this year. It was also stated that they had never previously considered free surfaces. Therefore, the NACHOS II code must not have this capability. It was also stated that they are looking at ProCAST and possibly other codes.

The proposed investment casting consortium would be organized similarly to the Specialty Materials Processing Consortium (SMPC) consortium, where Sandia directs the technical program and participates in the technical activities. What research efforts are to be funded and where they are to be performed is decided by the consortium. The results of the research efforts are available only to members of the consortium. The cost for each member will be \$50,000/year with DOE providing matching amounts.

Several general comments from the attendees are synopsized as follows:

- It seems like a worthwhile effort, well worth the investment.
- Modeling efforts to date are fragmented; have fallen short of expectations and need to be coordinated.

- Solid modeling capabilities need enhancement.
- Rapid prototyping, when functional, can significantly reduce costs.
- Industry needs to work together cooperatively to be competitive with Europe and the Pacific Rim.
- A great need exists for user support after the code is developed and in production.
- The analysis process must be robust, must work every time, with little to no problems.

It would seem reasonable, if the consortium is formed, to coordinate any MSFC CFD code development with the FASTCAST Process being developed at Sandia for a number of reasons:

- Sandia has had little to no effort in the fluid flow modeling task.
- The only Sandia code mentioned for consideration is NACHOS II, which is an old research code (not a production code) which will require significant modification, especially since it will not model free surfaces.
- The other code mentioned for consideration, ProCAST, is being evaluated/developed currently, but there is the question of proprietary rights and how the code could be used if chosen for FASTCAST.
- MSFC apparently has much more CFD expertise than Sandia.
- Developing a CFD code as a part of FASTCAST would possibly provide for instant acceptance of the code by the members of the consortium. Outside the consortium?... Maybe the consortium will sell or lease rights?
- A large negative might be in being forced to adapt Sandia interfaces for the code's input and output.

However, as a follow-up to the Sandia consortium meeting of November 5 - 6, 1991, a call was placed to see what progress was being made. They had not yet re-contacted the participants to establish a firm interest for the consortium. Apparently, the effort continues mostly as an in-house activity at this time.

NIST Consortium

Another consortium is being organized by the National Institute for Standards and Technology (NIST) and the National Center for Advanced Technologies (NCAT). A meeting was held January 16, 1992 at the Aerospace Industries Association (AIA) to determine if there is sufficient interest to form the Casting of Aerospace Alloys Consortium to improve the modeling process and perhaps develop a data base for selected alloys. Dr. Thomas Tom, Director, Advanced Technology, Howmet, sent me a

copy of his invitation including an invitation list. The list of 19 people included academia, engine builders, casting houses, a NIST representative, and a NSF representative. Note the obvious absence of NASA, government laboratories, and military organizations. However, according to the agenda, DARPA will be represented. An invitation to the meeting was acquired through a call to NCAT. The actual attendance list is reproduced in Table II.

The purpose of the consortium is to execute joint research efforts to develop process modeling tools and improve existing casting processes. NIST is to administer the consortium with Dr. Thomas (Tom) Yolken acting as Consortium Manager and Dr. William (Bill) Boettinger as Principal Scientist. Membership fees of \$10 - 15,000 will be used for administration of the consortium. At present, funding for industrial efforts will be provided solely by participants. DOD and other fundings will be sought. University fundings have also not been identified, but NSF and industry will try to help. It has been proposed that this consortium coordinate efforts with the DARPA consortium; however, the DARPA consortium membership is much more limited, and the exact mechanism of cooperation is not yet understood.

All the participants of the meeting were enthusiastic about the prospects of the consortium. Some of the industry representatives are already suggesting related joint efforts with competitors to pursue DOD and other funds. A group of volunteers were assigned the duty of contacting other industry representatives and establishing further interest and a consolidated consortium plan (better definition of research efforts). The results will be reviewed by the meeting participants and then sent out to all AIA member companies. Even if only a small percentage sign up, this could become a relatively large consortium.

A short time after the meeting, Dr. Thomas Yolken sent me a draft of the proposal for the consortium. After reviewing it, I called him with several comments and questions. We discussed some mechanisms by which NASA (particularly MSFC) can become a participating member of the consortium. Looking beyond the current evaluation/benchmarking of the ProCAST code, MSFC efforts of the Metallurgy Research Branch/EH23 in the areas of metallurgical thermophysical properties research and evaluation (and possibly others) can be used as "in-kind" research toward participation in the consortium. This and other topics in the consortium proposal have been discussed with Dr. Biliyar Bhat/EH23, who has shown interest in participating in the consortium and attended the second planning meeting April 28, 1992 in Gaithersburg, Maryland. Sign-up of members and initiation of the consortium research programs has now begun.

Other Consortia

Conversely, the DARPA Consortium to Develop Software for Solidification Modeling (called the Precision Investment Castings Consortium) currently will have only seven (7) members. These seven were chosen from an original group of 121,

which were reduced to 50, then to seven. The seven include HOWMET, Pratt & Whitney, PCC, UES, Allison, GE, and TiLine. Six million dollars will be funded by DARPA over a two year period. The members will provide matching funds. This consortium is also considered as an experiment in procurement and is expected to receive high level Congressional review. The consortium will be looking at generic technologies for casting: electronic data transfer, automatic mesh generation, automatic shell generation, material properties, etc. The goal is to develop technologies to achieve model preparation within 8 hours — currently it takes 1 - 4 weeks.

3.0. Evaluation of Codes

During the literature search, code search, consortium activities and discussions with industry personnel, much specific and general information was acquired concerning the available casting simulation codes. Based upon this information, which is certainly not exhaustive, the codes were evaluated for suitability of further development based on the following general criteria:

1. Does it contain a rigorous CFD mold filling model?
2. Does it contain a good solidification model?
3. Does it contain a good cooldown model?
4. Is the simulation relatively fast and accurate?
5. Is it relatively user friendly and inexpensive to use?
6. Is it supported by the developers?
7. Is it continually being improved?

While most of the codes in Table I either do not contain mold filling capabilities or are not generally used in the casting industry (for reasons described earlier), only a few codes reasonably satisfy all the above criteria:

- ProCAST
- RaPiDcast
- Magmasoft
- Simulor

Any one of these four would be a good candidate for further development. However, ProCAST was chosen as the best candidate for the following reasons. While both RaPiDcast and ProCAST are essentially equivalently ranked on criteria 2 through 7, ProCAST has the advantage in terms of a rigorous CFD mold filling model primarily because the solution algorithms are formulated using finite element methods rather than finite volume (finite difference). Finite element formulations are generally more accurate than finite difference methods and represent the preferable approach for accurate simulations of complex investment castings.

Magmasoft is reportedly the strongest competition to ProCAST, and it also uses a finite difference formulation. But since very little information has been located in the open literature, probably due in part to its development and marketing by a German firm (Magma), the code was not selected for further development in the current effort.

The Simulor code, developed and marketed by a French firm (Aluminium Pechiney), uses a finite difference approach, but does include some useful options to reduce execution times at the expense of accuracy. Simulor suffers from the same lack of exposure in the open literature, and was also eliminated from further development in the current effort.

Since the evaluation of codes was not strictly objective (i.e., reduced to a numerical comparison), it should again be stated that the selection of ProCAST as the best candidate for further development was not simply derived by the above comparisons or earlier stated comparisons, but was additionally influenced by communications with industry personnel.

At this point, a decision must be made as to whether the ProCAST code will be further developed or a new code developed. The development of a new code, including documentation, benchmarking, etc. would be very costly and there are the questions of user support and continued development. The ProCAST code represents a very good basis with well chosen mathematical models, on-going development, and continuous user support. A duplication of any of this development effort seems unwarranted and self serving. Therefore, a decision was made to help develop the ProCAST code via benchmarking the code's capabilities.

4.0. Benchmarking of ProCAST Code

Further development of the ProCAST code via benchmark analyses on the MSFC ED32 Silicon Graphics system was agreed upon with UES whereby MSFC would be allowed free use of the code on a monthly basis for an undetermined length of time. UES sent to MSFC an executable copy of the latest version of the code with mold filling capabilities (version 2.0), and later updated this with versions 2.0.2. and 2.1.1. They also sent a User's Manual for version 2.0, and later updated it with a version 2.1 manual. Review of the manual revealed much about the use and structure of the code. The code is modularized with the file structure arranged for easy communication between modules. There are actually five separate modules that may be executed for a complete analysis. A brief description of the modules and their functions and capabilities follows:

PreCAST is used to completely define the analysis. The finite element model can be imported from PATRAN, IDEAS, or ANVIL. Or if the geometry is only 2-D, it can be generated with a CAD-type module (called CREATE-2D). Boundary conditions, material properties, heat transfer data and run-time parameters are also specified in PreCAST.

DataCAST reviews the total model and performs extensive error checking. All units are converted to cgs and a summary file is created for the complete analysis model. This file should be manually inspected before going on to ProCAST.

ProCAST performs the simulation analysis. It contains all the mathematical modeling and solution algorithms and techniques.

PostCAST is used to generate postprocessing files that can be viewed using PATRAN, IDEAS or ViewCAST.

ViewCAST generates a wide variety of graphical representations of the simulation solution.

During the course of the benchmark analyses, several communications were made to UES, the developers of the ProCAST code. They were of much assistance in resolving difficulties in understanding the code usage. In some cases, it was a misinterpretation; in others, there were differences between the code and the manual, and others were simply minor errors in the functions of the interactive operations (inherent in a developing code). In addition to resolving these problems, they have been very receptive to suggestions on possible improvements in specific operations of the code. Some of these suggestions and many other code improvements have been incorporated in updated versions (2.0.2 and 2.1.0) of the code, which were received and installed for use during the course of this effort.

Two benchmark cases and a mold filling demonstration were completed in the current effort. The two benchmark cases are simulations of experimental configurations where much test data is available for direct comparisons with predicted results. While the two cases do not involve actual mold filling transients, they both represent steady state conditions which can exist behind the free surface during mold filling, and the use of liquid metal as the fluid media is not necessary since the code has the flexibility to model any common fluid, liquid, or gas.

The backward-facing step case (Section 4.1) involves air flowing through a duct with a sudden increase in flow area. The flow inherently separates from the wall at the discontinuous (step) surface and re-attaches to the wall further downstream. Additionally, a series of adverse pressure gradients are formed downstream of the re-attachment, on the opposite wall and possibly within the primary separation region. These gradients, if strong enough, produce additional separation (recirculation) regions. The ProCAST code's ability to predict the existence and locations of these regions was tested in this case. The results are compared to both test data and predictions from Reference 149.

The duct flow case (Section 4.2) involves water flowing through a square duct turning a 90° bend. Higher pressures are generated on the outside of the turn than on the inside, creating a crossflow pressure gradient. This affects a secondary flow with radial and spanwise components which significantly affect the streamwise velocity profiles. The ability of the ProCAST code to predict the complete three-dimensional velocity profile was tested, and the results are compared with both test data and predicted results from Reference 150.

Other benchmark cases were in various stages of completion when the efforts were terminated due to lack of funds. These cases will be briefly discussed in Section 4.3 along with an earlier mold filling demonstration which was performed simply for qualitative evaluation of the code's ability to track a free surface undergoing extreme distortions.

4.1 Backward-Facing Step

The two-dimensional backward-facing step geometrical characteristics are depicted in Figure 1. Fully-developed laminar flow enters from the left side and encounters an instantaneous (step) increase in flow area. The flow detaches at the corner and reattaches to the lower wall at location X_1 . Two other experimentally determined separation (recirculation) regions can occur as located by X_2 & X_3 and X_4 & X_5 . Figure 2 (extracted from Reference 149) depicts the locations of these regions for air at various Reynolds numbers in terms of multiples of the step height, S . The predictions performed in Reference 149 (Figure 3) indicate an additional separation region, located by X_6 and X_7 , within the primary separation region (upstream of X_1) for $Re > 1000$.

The model for this case was chosen to be two dimensional along the centerline of the apparatus, ignoring spanwise variations and effects. Although the experimental apparatus was necessarily three dimensional, the composite data of Reference 149 shows that 3-D effects are minimal below $Re = 400$. A 2-D model was chosen primarily for two reasons: 1) Simplicity, since this was the first modeling attempt with the code, and 2) To use the code's mesh generation capability, which utilizes only 2-D triangular elements.

Three different fluid mesh sizes (Figure 4) were used to model the step flow. The coarse grid has only five freestream nodes across the height of the downstream channel, whereas the medium mesh has eleven and the fine mesh has twenty. The meshes in Figure 4 depict only a small portion of the entire model, whereas the step height, S , is 0.49 cm and the model extends upstream for 20 cm and downstream for 20 cm. Comparing the total number of elements and nodes shows the vast differences in sizes of the three models.

	<u>No. of elements</u>	<u>No. of nodes</u>
Coarse mesh	2,733	1,836
Medium mesh	9,575	5,261
Fine mesh	26,997	14,291

The Dirichlet boundary conditions applied to these models include:

1. No slip at the walls, $u=v=0$, standard conditions for a viscous solution.
2. Inlet velocity: horizontal component only, vertical component is zero. The average magnitude of the inlet velocity varies with desired Reynolds number as described below.
3. Exit pressure: One atmosphere applied across the exit plane.

Reynolds numbers for each case were determined using a technique compatible with that of Reference 149:

$$Re = \rho V D / \mu,$$

where, for air at 1 atm and 20 °C,

$$\rho = 0.001204 \text{ gm/cm}^3$$

$$\mu = 1.824 \text{ E-5 N sec/m}^2$$

The characteristic dimension D is computed as the hydraulic diameter of the inlet channel and is equal to twice its height, $D = 2h$. The velocity, V, used in Reference 149 was defined as two-thirds of the measured maximum inlet velocity, which corresponds in the fully-developed laminar case to the average inlet velocity. The actual value of the input inlet velocity used in each model is dependent on the mesh configuration since a linear interpolation of the Dirichlet velocity boundary condition is used between the wall and the adjacent node. For a uniform velocity applied to all freestream nodes across the inlet plane, this results in different average velocities as depicted in Figure 5. When a uniform velocity was used across the inlet freestream nodes, the following table defines the velocities (in cm/sec) required for each Reynolds number used in the parametric analyses.

Re	Required Average Velocity	Velocity (Coarse Mesh)	Velocity (Medium Mesh)	Velocity (Fine Mesh)
100	14.67	22.0	18.3	16.5
1000	146.7	220.0	-	-
1200	174.8	-	220.0	196.7
2000	293.4	440.0	-	-
2400	349.6	-	440.0	-
5000	733.5	1100.0	-	-
6000	880.0	-	1100.0	-

While the above input velocity profiles do not accurately represent developed flow, they are simpler to input and represent an insignificant source of error for the current models. The length-to-height ratio (l/d) of the entrance duct is 38.5 (20 cm/.52 cm), more than sufficient for the velocity profile to fully develop before reaching the step. In fact, some of the medium and fine mesh models were re-run with a better representation of fully developed laminar input velocity profiles. This resulted in no discernible differences in the flow field solutions at and downstream of the step.

Converged solutions for all cases were obtained through a series of analyses utilizing the code's restart capabilities. Since these analyses required steady state

solutions to a relatively complex flow field filled with flow detachments and reattachments (regions of separated flow), the solution parameters were changed before each restart to ensure that a progressively more accurate solution was obtained. This is different from a normal casting simulation, which is transient, with the dominant concern being the location of the liquid metal free surface. The prediction of the flow field behind the free surface is also important for prediction of mold heating and metal cooling and solidification. So while the current analyses represent a solution technique somewhat different from a normal casting, it is important that the results be reasonably accurate. Also, these more accurate solutions require additional iterations and CPU time beyond what would be used for a typical mold filling simulation. The iteration and time requirements are given below for comparison purposes only, and even so, are not directly comparable because of the lack of an objective numerical criteria for convergence of steady state solutions. As is always the case, the finer meshes require more setup time, data storage requirements, CPU time, and data reduction time, and they are more difficult to achieve convergence. Convergence was subjectively determined from observations of on-screen graphics, usually sequential x-velocity contours at every twentieth iteration. When the changes became very small or, in some cases, exhibited small oscillations about a stationary norm, the case was re-run with an additional 100 to 200 iterations to ensure that no additional change in the solution occurred.

Re	Coarse Mesh (iter/CPU-hr)	Medium Mesh (iter/CPU-hr)	Fine Mesh (iter/CPU-hr)
100	400/0.3	1600/5.9	3100/43.9
1000	600/0.4	-	-
1200	-	600/3.5	2100/35.5
2000	1000/0.6	-	-
2400	-	800/3.3	-
5000	400/0.3	-	-
6000	-	400/1.8	-

Post-processing of the solutions was initiated with the examination of the entire mesh network to ensure that the elements and nodes were located properly and that boundary and initial conditions were applied correctly. Also, the initial determination of detachment and reattachment points was made by examining the velocity components of the fluid nodes immediately adjacent to the walls. It became immediately obvious that the location of flow reversal (in the longitudinal, x-direction) was not the proper location of the separation/reattachment since the slope of the zero-velocity line was very shallow and the mesh definition was not very good, particularly for the coarse mesh. Interpolation methods would be needed to accurately project the zero-velocity line to the wall. These methods are already available in ViewCAST (the post-processor in ProCAST), where velocity vector and velocity contour plots can be used to better examine the entire flow field.

An example of a velocity vector plot is shown in Figure 6, which represents a portion of the backward-facing step flow field solution for the medium mesh with a Reynolds number of 100. This type of plot portrays a global picture of the flow field. While some features are readily discernible, such as the primary separation aft of the step and changes in velocity vectors indicated by changes in direction/size/color of arrows, quantitative results are not readily obtained directly from the figure. The recirculation of gases in the primary separation zone is apparent, but locating the exact boundary of the primary zone would be difficult. For this type of information, a velocity contour plot would be much more meaningful.

Figure 7 is a representative velocity contour plot of the x-component of velocity for the fine mesh with Reynolds number of 1,200. The typical color spectrum has been modified to provide a better contrast between color (x-velocity magnitude) changes. Note that the zero velocity value occurs at the interface between blue and yellow. This interface (or zero-velocity line) is rather well-behaved and essentially linear down to the last set of freestream nodes adjacent to the wall. The contour plotting interpolation routine breaks down at this point and cannot correctly project the line onto the wall. This is a result of a change in sign of the x-component of velocity (flow reversal) between two freestream nodes adjacent to the no-slip wall nodes. The contour code cannot project the zero-velocity line beyond this location.

The projection of the zero-velocity line to the wall is the same technique used in the experimental data of Reference 149 to locate attachment and detachment locations. However, the non-intrusive laser-Doppler anemometer used in Reference 149 allowed measurements very close to the wall so that the projection inaccuracies were minimized. Measurements within 0.1 mm of the wall were routinely made, providing accurate locations of the long, shallow separated flow regions.

Table III shows the separation regions that have been reduced from the current analyses. All of the primary separation regions (terminated at X_1/S) have been predicted. Note that only the re-attachment of the primary separation region (X_1/S) is identifiable with the coarse mesh solution. Sufficient mesh density is not available adjacent to the walls (nodes approximately 0.20 cm from the walls) to detect the shallow separation zones that are present above a Reynolds number of 400. The medium mesh, with about four times as many elements, has a better definition at the walls (nodes approximately 0.09 cm from the walls), and is able to compute some separation regions and provide strong indications of others. Data from the fine mesh solutions with its higher density mesh (nodes approximately 0.055 cm from the walls) better defines all separation regions for the two Reynolds numbers (100 and 1200). Some of the top wall regions, defined by X_4/S and X_5/S , have been located and projected to the wall. Some of the top wall data indicate sharp drops in the x-component of velocity but no flow reversal. This indicates the presence of an adverse pressure gradient and the possibility of a separation region that is shallower than can be computed by the mesh size. A better definition (denser mesh) at the walls is normally provided with fewer nodes by packing the nodes near the walls, but this

capability is not available with the CAD-type CREATE-2D option used for this model. Furthermore, while the coarse and medium meshes do not provide extreme accuracy, they are probably adequate for most casting simulations.

Figure 8 shows the only valid comparison that can be made with test data (Reference 149) since the $Re = 100$ case is the only one below the 3-D effects, which start at $Re = 400$. Reference 149 states that: ".....with the occurrence of more than one separated flow region, the flow in the experiments becomes three-dimensional in the region downstream of the step, and this prevents direct comparison between the experimental and theoretical results." Note in Table III that as the mesh gets finer, the prediction of primary re-attachment, X_1/S , for the $Re = 100$ case increases from 1.9 to 3.0, with the 3.0 being the only one represented in Figure 8. In agreement with test data, only the primary re-attachment is predicted, and the accuracy is very good. The fine mesh solution did show indications of a near separation region on the top wall in the form of a significant increase in boundary layer thickness. At higher Reynolds numbers a separation should occur downstream of this vicinity. And, if the X_4/S and X_5/S test data is closely examined, the curve fit of the data could easily be extended to lower Reynolds numbers at further upstream locations.

Figure 9 compares the fine mesh solutions with the predictions of Reference 149, which are steady state 2-D solutions, making them more directly comparable to the current predictions above $Re = 400$. Since the Reference 149 predictions are laminar, they are not valid beyond the experimentally determined turbulent transitional region at $Re = 1250$. While the $Re = 100$ case matches almost identically, the $Re = 1,200$ case does not. The current fine mesh solution predicts a smaller primary recirculation region and a larger top wall separation region that does the prediction of Reference 149. However, which one is more accurate is not determinable.

An anomaly occurred in the exit region of the $Re = 100$ cases, but it is improbable that this had any significant effects on the flowfield solutions near the step region. The anomaly was very localized at the exit and appeared to be caused by a boundary layer separation from the wall. Since only a pressure boundary condition is applied at the exit plane, the reverse flow in the separated region is supplied by flow entering through the exit plane as long as the specified pressure is satisfied. See the anomaly discussion at the end of Section 4.2 for the solution to this problem. This phenomenon did not occur at higher Reynolds numbers.

4.2 Duct Flow

This benchmark case consists of water flowing through a constant area duct of square cross section (40 mm x 40 mm) turning a 90° bend with a 2.3 radius ratio and a Reynolds number* of 790, corresponding to a Dean number† of 368. Two 3-D finite element meshes shown in Figures 10 and 11 were generated using PATRAN. The first mesh is 7x7x67, consisting of 7x7 equal spaced nodes at each of 67 cross sections, resulting in 2,376 brick elements with 3,283 nodes. The second mesh is 13x17x61, with nodes packed at the walls, shortened entrance and exit lengths and utilizing symmetry conditions across the span, resulting in 4,320 brick elements with 5,551 nodes. Both of these represent much coarser meshes than the 21x21x51 mesh used in Reference 150, as depicted in Figure 12. The solutions in the reference were obtained using a CRAY XMP, which obviously has much more computing power than the SGI Personal Iris workstation used in this effort.

Boundary conditions consisted of a pressure of one atmosphere across the exit plane, no slip ($u=v=w=0$) at the walls, and an inlet velocity profile derived from test data. Figure 13a depicts the test data velocity contours and the smoothed contours (normalized to 1.98 cm/sec) established in Reference 150, with the 7x7 uniform grid superimposed. The inlet velocity profile was taken directly from the mesh overlay. The actual input velocity map (in cm/sec) is shown in Figure 13b, where, for the inlet plane only, the velocities are interpolated linearly between all nodes. Figure 14 shows the packed mesh inlet velocity profiles as interpolated by the ProCAST code. While interpolation in the core region (where velocity gradients are small) appears distorted, actually it accurately represents the input velocities and produces no significant computational deficiency.

The steady state solution to the uniform mesh case was obtained with 1000 iterations requiring 20 hours of CPU time on the SGI system. The solution of the packed mesh case required 1800 iterations and 28 hours of CPU. The interpolation of the results was accomplished through a series of velocity contour maps (generated using ViewCAST) at various planes within the flowfields. Figures 15 and 16 show velocity contour maps depicting velocity magnitudes around the bend and at a

* Reynolds number is defined as:

$$Re \equiv \rho v d / \mu$$

where ρ is water density

v is average inlet velocity (1.98 cm/sec)

d is hydraulic diameter (40 mm)

μ is water absolute viscosity

† Dean number is defined as:

$$De \equiv Re \sqrt{d / (r_i + r_o)}$$

where Re is Reynolds number

d is hydraulic diameter (40 mm)

r_i is inside radius of bend (72 mm)

r_o is outside radius of bend (112 mm)

downstream cross-section of $0.25d$ for both uniform and packed mesh cases. The contour maps around the bend (water flow from right to left) depict the velocity profiles along the symmetry plane (the center of the duct in the spanwise direction, z). While the contrasting color spectrums make it more difficult to peruse the global environments, it is very advantageous when determining velocity profiles at a given location, especially since velocity profiles are not directly obtainable from ViewCAST. The inset contours represent cross-sections as indicated from a location $0.25d$ downstream of the end of the bend, where d is the hydraulic diameter, 40 mm. Note that both sides (mirror images) of the symmetry plane are shown in Figure 15, but are not in Figure 16, where only one side of the duct flow is computed.

When a fluid is turned by a duct, the induced centrifugal forces and the frictional effects at the walls combine to create a secondary flow in and downstream of the bend. The centrifugal forces decelerate the flow on the outside (pressure) surface, resulting in increased pressure and a crossflow pressure gradient toward the inner (suction) surface. The frictional effects (creating the boundary layer) provide a path for the pressure gradient to produce a secondary flow consisting of two counter-rotating vortices. Figure 17 shows this at the $+0.25d$ plane in the form of y -velocity contours. The flow along the side walls is produced by the cross-flow pressure gradient in the boundary layer, while the flow in the core region is in the opposite direction.

The velocity profiles deduced from the color velocity contour plots are compared in Figures 18 and 19 to test data and predicted results from Reference 150. The plots for each location contain six sets of data: the test data is represented by symbols (open circles), the predictions from Reference 150 are represented by a dotted line for the 21×21 mesh, a short dashed line for the 31×31 mesh, and a solid line for the 41×41 mesh, and the current predictions are represented by long dashed lines. Figure 18 represents streamwise velocity profiles radially along the plane of symmetry at four different locations around and downstream of the bend as indicated, where $\theta=0$ represents the start of the bend. The three predictions of Reference 150 match the test data fairly well, with the coarsest mesh beginning to deviate from the other two. The current predictions do not match as well, but the packed mesh does much better than the uniform mesh. The same trend is true for the profiles of Figure 19, where spanwise (z -direction) profiles of streamwise and radial velocity components are depicted at the normalized radius $R^*=0.2$, which corresponds to 12 mm from the inside (suction) surface. It is conceivable that if the packed mesh grid density was increased, the prediction would continue to approach the test data and/or other predictions, but because of machine limitations, this is not possible.

Two factors, both relating to the coarseness of the mesh, are believed to be responsible for the extreme inaccuracies of the uniform mesh solution. First, the inlet velocity profile (Figure 13b) does not accurately represent the boundary layer. Note that a linear interpolation from the wall to the first freestream node (6.67 mm from the wall) provides velocities well below test data values for the entire 6.67 mm, thus effectively providing a much thicker boundary layer. Secondly, since there are no

nodes within the boundary layer, which with this geometry represents more than half the cross-sectional area, the strong secondary flow that originates here cannot be predicted accurately. Further, this lack of definition denies an accurate prediction of an adverse streamwise pressure gradient along the suction surface and a favorable gradient along the pressure surface, which significantly affect the development of the streamwise velocities. Moreover, the basic problem is that the oversized boundary layer produces too much core flow so that the effects of the secondary flow and streamwise pressure gradients are subdued. The thicker than desired boundary layers are easily seen in all profiles of Figures 18 (wall at $R^*=-0.5$) and 19 (wall at $z^*=-0.5$). The nodal locations for the 7x7 mesh are superimposed on the right side of each of the figures for reference. It is obvious that additional nodes in the boundary layer are needed if a better accuracy is to be achieved. This is normally accomplished by using a finer mesh and packing the nodes near the wall, as done in Reference 150 (Figure 11a) and the packed mesh case.

The packed mesh analysis much better predicts the overall flow structure primarily because of the increased nodal density within the boundary layer. Where the uniform mesh predicted a much too thick boundary layer, the packed mesh predicts, in many locations, less boundary layer than any of the Ref. 150 predictions. The experimental data does not adequately represent any boundary layers. Since the Ref. 150 predictions generally provide a much better fit of the experimental data, it is assumed that the packed mesh predicted boundary layer is truly too thin. Again, a higher mesh density will probably improve the predicted results.

An anomaly very similar to that described in Section 4.1 again occurred in the exit duct of the uniform mesh case. The exit duct is very long with an l/d of 50, and flow separation occurred at an l/d of approximately 30. Reverse flow in the separated region resulted in reverse flow through a portion of the exit plane, just as was the case for the backward-facing step. Also, just as before, it should not cause any inaccuracies of the flow solution in the vicinity of the bend since it occurred so far downstream. It was later determined by UES personnel that the anomaly was created by an incompatible set of boundary conditions and model orientation. Applying a constant pressure across the exit plane, which is parallel to the gravity vector, creates an inconsistency which dominates the solution at low Reynolds numbers. A gravitational gradient at the exit plane cannot be computed by the code since the uniform boundary condition must be satisfied. The problem was eliminated by using no gravity vector (zero gravity). Since the model involves water flowing horizontally with no significant heights, the accuracy of the solution will not suffer. Other models have been used with similarly inconsistent boundary conditions which created similar effects in the exit region at low Reynolds numbers. However, those models contained sufficient exit lengths (away from the flow of interest) so that the anomalies could be ignored.

4.3 Volume Filling, Turbulence, and Waves

A simple 2-D mold filling model was formulated to qualitatively evaluate the ProCAST code's ability to track the liquid metal free surface during the mold filling transient. This model involves no heat transfer (i.e., all surfaces are adiabatic) but does demonstrate many fluid flow phenomenon. A finite element mesh of 5,103 triangular elements and 3,016 nodes with molten iron flowing in at a rate of 20 cm/sec used 300 time steps to predict the 6-second filling transient, requiring approximately one hour of CPU time. Eight sequenced snapshots showing the iron filling the mold cavity (mold not shown) are depicted in Figure 20. The color contours represent velocity magnitudes, independent of direction. Several intuitive observations can be made:

- In Figure 20a, the horizontal arm is being filled by horizontal convection driven by potential energy (differences in free surface heights).
- In Figure 20b, the horizontal arm and column are filled and the flow down the ramp is again the result of gravity converting potential energy into kinetic energy. Note that because of the fluid viscosity, the highest velocities are achieved on the liquid surface.
- The momentum of the fluid in Figure 20c causes it to follow the circular surface of the mold creating a crest which, afterward, does free fall back to the higher velocity surface. Note also that velocities exceeding 100 cm/sec are achieved at the surface of the fluid near the bottom of the ramp.
- After the wave falls in Figure 20d, a side-to-side sloshing motion is established and persists throughout the filling transient. Depending on the position and motion of the sloshing fluid, the high velocity fluid flowing down the ramp either penetrates the sloshing fluid (Figure 20f), flows along the top of the sloshing fluid (Figure 20g), or is in a transition between these two conditions (Figures 20e and 20h). Examination of the entire recorded solution (every 10 steps) much better reveals the transition between these events.

While no experimental data are available for verification and no quantitative assessment has been accomplished, the motion of the fluid appears as expected.

Both 2-D and 3-D models were used to examine the code's ability to accurately compute known volumes and smoothly fill these with time accuracy. The 2-D model was a 10 cm wide, 200 cm tall box, filled from the bottom with a 20 cm/sec flow of inviscid water (with slip conditions at the walls). The predicted filling time was 10.005 seconds (versus an exact time of 10.0 seconds), and the free surface remained relatively smooth. The 3-D models consisted of the uniform mesh duct model rotated 90° so that the exit length is vertical. Inviscid water flowing at 20 cm/sec fills the duct from the bottom. The free surface is not smooth because of the bench, and the filling time is computed to be 11.977 seconds versus an exact time of 11.518 seconds. This

is probably due in part to the computed volume being 3,910.1 cm³ versus 3,685.8 cm³ actual. This discrepancy in volumes was larger than expected and was reported to UES. They are in the process of examining these differences.

A 2-D turbulent model of a backward-facing step (Ref. 152) is being modeled to examine the predictive capability of a turbulent shear layer. The geometry is similar to the previous step model examined except with a much lower expansion ratio and the absence of 3-D effects at high Reynolds numbers in the region of interest. To examine the effect of turbulence, both a non-turbulent and a turbulent solution were to be run. However, only the turbulent solution was to be compared to test data and previous predictions. Four models of the 2-D turbulent backward-facing step were attempted with various mesh sizes. Node counts of 13,229, 11,651, 8,760, and 4,954 were all tried in an attempt to accomplish a solution. The largest model exceeded the machine's capacity. The second largest would run for only 52 iterations before exceeding the capacity. The two smaller models would run for approximately 150 iterations in the laminar mode, but would not start to execute in the turbulent mode. Discussions with UES determined the need for a later version of the code which contains an improved turbulence model. The new code was received and installed. The laminar case was run for 1,000 iterations, with the converged results depicted in Figure 21. The turbulent case still would not run satisfactorily and UES is currently investigating this problem.

A 1-D model of a Burger wave (Ref. 153) was formulated and successfully executed with an earlier version of the code. The reduction of data and comparison to other numerical solutions was delayed during the recent problems with the turbulence cases. Plans were to re-run it with the new code and then start the data reduction. This was never completed.

5.0 Conclusions

Several casting simulation codes contain mold filling capabilities, but only a few contain the fluid dynamics sophistication desired. Of these codes, ProCAST was chosen as the best candidate for further development via benchmark analyses. The decision was made that this approach would be better than developing an entirely new code since an enormous effort has already been expended on ProCAST and the modeling approach would be very similar. Furthermore, the level of continued development and user support provided by UES, Inc. (developer and marketer of the ProCAST code) cannot easily be matched – and should not be. Therefore, the most cost effective approach was to help the casting industry evaluate the ProCAST code.

The results of the two benchmark cases show that the code can accurately predict certain steady state 2-D and 3-D laminar flow fields if the finite element mesh size is small enough. When the mesh size is increased, the accuracy of the flow details is reduced, as expected. Knowing how the solution will be affected by a larger grid size is an important feature when typical casting simulations are performed with a minimal number of fluid elements.

6.0 Recommendations

In order to more thoroughly benchmark the ProCAST code so that it can be used as an analysis tool to support casting and quality issues, it is desirable to extend the current effort to include the following:

- Continued development of the ProCAST code via additional benchmark cases to include:
 - Complete the 2-D pressure wave/reflection cases to examine the dissipation effects and time accuracy of the code,
 - Complete the turbulent case since turbulence is the source of many defects, particularly inclusions of oxides sheared from the mold surfaces,
 - A 2-D Howmet mold filling case to model liquid metal free surface movement and the creation of hot spots on the surface and subsurface of the mold,
 - A 3-D solidification model of an SSME part to evaluate all aspects of the code, including macro and micro modeling capabilities.
- Update the literature and code reviews, placing more emphasis on solidification modeling.

Table I. Survey of Casting Simulation Codes

Code	Developing Organization	Mold Filling Attributes	Solidification Attributes	Hardware Req.	Cost K\$	Ref. #
AFSolid	American Foundryman's Society	X	2-D with variable thickness. Predicts shrinkage. Can input hot spots. Good pre- and post-processing.	PC	2	57, 58
C.A.S.T	Abex Corp.	X	F. D.	•	•	61, 62
CADCAST	NSF/UAT, GIT, UM	•	•	•	•	62
COMMIX	Argonne N. L.	•	•	•	•	•
Duct/Moldflow	Cambridge Univ.	Generates F. E. grid for flow and stress analyses.	•	•	•	56, 62
FASTCAST		Not available yet.	•	WS or Super	•	•
FIDAP		Free surface capability.	•	•	•	•
FLOCAST	Univ. of Pittsburgh	Uses SOLA-VOF.	•	MINI or WS	•	12
FLOW 3-D	FLOWSCIENCE/ Los Alamos N. L.	F. D. Long run times. Not user friendly, but good numerical results. Includes turbulence model.	•	•	•	8, 9

Notes: 1. All accept CAD input unless otherwise noted.

- Incomplete information
- No capability: not available
- 2. All codes have 3-D capability except for AFSolid which is 2-D but has variable thickness capability.

Table I. Survey of Casting Simulation Codes, continued

Code	Developing Organization	Mold Filling Attributes	Solidification Attributes	Hardware Req.	Cost K\$	Ref. #
Magmasoft	Magma, Germany	F. D.	Structure formation and hot tears	MINI	70+	57
MSS	Univ. of Michigan	X	F. E.	•	•	56
NEKTON	Nektonics/Creare.x	F. E. Uses spectral element method. Full unsteady N. S. solution. Solves conjugate heat transfer.	X	WS or Super	•	40
NOVACAST	Nova, Sweden	X	Modulus based	•	•	56
Phoenix	CHAM of N. A.	Includes chemical reactions and combustion processes. EasyFlow is PC counterpart.	•	WS or SUPER	8-14/yr	57
ProCAST	UES, Inc.	F. E. Full N. S. solution. VOF approach. PreCast interfaces with PATRAN, IDEAS and ANVIL.	Uses enthalpy method. Contains alloy database. Moving-surface radiation view factors	MINI, WS, or SUPER	5-50/yr	1, 2, 3, 54, 57

Notes: 1. All accept CAD input unless otherwise noted.

- Incomplete Information
- X No capability: not available
- 2. All codes have 3-D capability except for AFSolid which is 2-D but has variable thickness capability.

Table I. Survey of Casting Simulation Codes, continued

Code	Developing Organization	Mold Filling Attributes	Solidification Attributes	Hardware Req.	Cost K\$	Ref. #
RaPiDcast	Metalworking Technology, Inc.	F. D. Based on SOLA-VOF. Includes turbulence model. Full unsteady N. S. solution.	No natural convection. Uses enthalpy method. Predicts thermal stresses.	•	•	38, 53
Simulor	Aluminium Pechiney	F. D. Time accurate or use a "false transient approach." Free surface smoothing option.	Uses enthalpy method.	WS	105	4, 5, 39, 57
SOLCAST	GM (Proprietary)	X	F. E. Tailored to GM use.	•	•	•
Solstar	Foseco	X	F. D. Versatile post processing	PC	60+4/yr	57, 69
SPIDER	Univ. of Illinois	X	F. E.	•	•	62
Swift	Univ. of Wisconsin	X	Modulus based	WS	15+3/yr	57, 62
TOPAZ	Lawrence Livermore Natl. Lab	X	Relatively slow, poor pre-processor	•	•	•

Notes: 1. All accept CAD input unless otherwise noted.

- Incomplete information
 - X No capability: not available
2. All codes have 3-D capability except for AFSolid which is 2-D but has variable thickness capability.

Table II. Attendees List
Casting of Aerospace Alloys Consortium Planning Meeting
January 16, 1992
Aerospace Industries Association, Washington, DC

1. Prof. John T. Berry, Univ. of Alabama, Dept. of Metallurgical & Materials Eng.
2. Mr. Michael Blaney, NIST
3. Dr. William Boettinger, NIST, Metallurgy Division
4. Dr. C. Robert Crowe, DARPA, Naval Research Laboratory
5. Dr. Ared Cezairliyan, NIST, Thermophysics Division
6. Prof. Jonathan A. Dantzig, University of Illinois, Dept. of Mech. & Ind. Eng.
7. Ms. Sara Dillich, Staff Engineer, U. S. Bureau of Mines
8. Mr. Robert Ehrenstrom, Allison Gas Turbine Division, General Motors Corp.
9. Mr. Donald J. Frasier, Allison Gas Turbine Division, General Motors Corp.
10. Mr. J. A. Friedericy, Dir., Research & Tech., Allied-Signal Aerospace Co.
11. Dr. Harold L. Gegel, Dir., Process Science Div., UES, Inc.
12. Dr. Larry Graham, Technical Director, PCC Airfoils, Inc.
13. Mr. Richard Hartke, AIA, Inc.
14. Dr. Sulekh C. Jain, GE Aircraft Engines
15. Dr. Bruce M. Kramer, Prog. Dir., Mat. Proc. & Mfg., Natl. Science Foundation
16. Dr. Bruce A. MacDonald, National Science Foundation, Div. Mat'l Research
17. Dr. Francois R. Mollard, Mgr., Casting Dept., Metalworking Technology, Inc.
18. Mr. Enrique E. Montero, United Technologies, Pratt & Whitney
19. Mr. Jesse Murph, ERC, Inc.
20. Dr. Bruce T. Murray, Applied & Computational Mathematics Div., NIST
21. Mr. Donald R. Parille, United Technologies, Pratt & Whitney
22. Dr. Anand J. Paul, Dir., Engineering Analysis, Metalworking Technology, Inc.
23. Dr. Thomas Piwonka, Dir., Metal Casting Tech. Center, The University of Alabama
24. Dr. Emanuel Sachs, Massachusetts Institute of Technology
25. Dr. Kim A. Stelson, Dept. of Mechanical Engineering, University of Minnesota
26. Mr. Dennis C. Stewart, United Technologies, Pratt & Whitney
27. Prof. Julian Szekely, Dept. of Mat'l Science & Eng., Mass. Institute of Tech.
28. Dr. Thomas Tom, Dir., Adv. Technology, Howmet Corporation
29. Prof. Kuo-King Wang, Sibley Prof. Mech. & Aerospace Eng., Cornell Univ.
30. Dr. H. Thomas Yolken, Chief, Office of Intelligent Processing & Materials, National Institute of Standards & Technology

Table III. Predicted Detachment and Re-Attachment Locations From Backward Facing Step Solutions

Coarse Mesh

Re	X ₁ /S	X ₂ /S	X ₃ /S	X ₄ /S	X ₅ /S	X ₆ /S	X ₇ /S
100	1.9	X	X	X	X	X	X
1,000	5.0	X	X	X	X	X	X
2,000	5.4	X	X	X	X	X	X
5,000	7.1	X	X	X	X	X	X

Medium Mesh

Re	X ₁ /S	X ₂ /S	X ₃ /S	X ₄ /S	X ₅ /S	X ₆ /S	X ₇ /S
100	2.6	X	X	X	X	X	X
1,200	6.3	X	X	(3.5)	(10.0)	X	X
2,400	7.3	X	X	6.3	10.1	[0]	[2.7]
6,000	6.1	X	X	6.1	11.6	X	X

Fine Mesh

Re	X ₁ /S	X ₂ /S	X ₃ /S	X ₄ /S	X ₅ /S	X ₆ /S	X ₇ /S
100	3.0	X	X	(3.8)	(4.7)	X	X
1,200	4.9	X	X	3.1	11.5	X	X

X No event detected.

() No separation, but significant increase in boundary layer thickness.

[] Indications are present, but flow not well established.

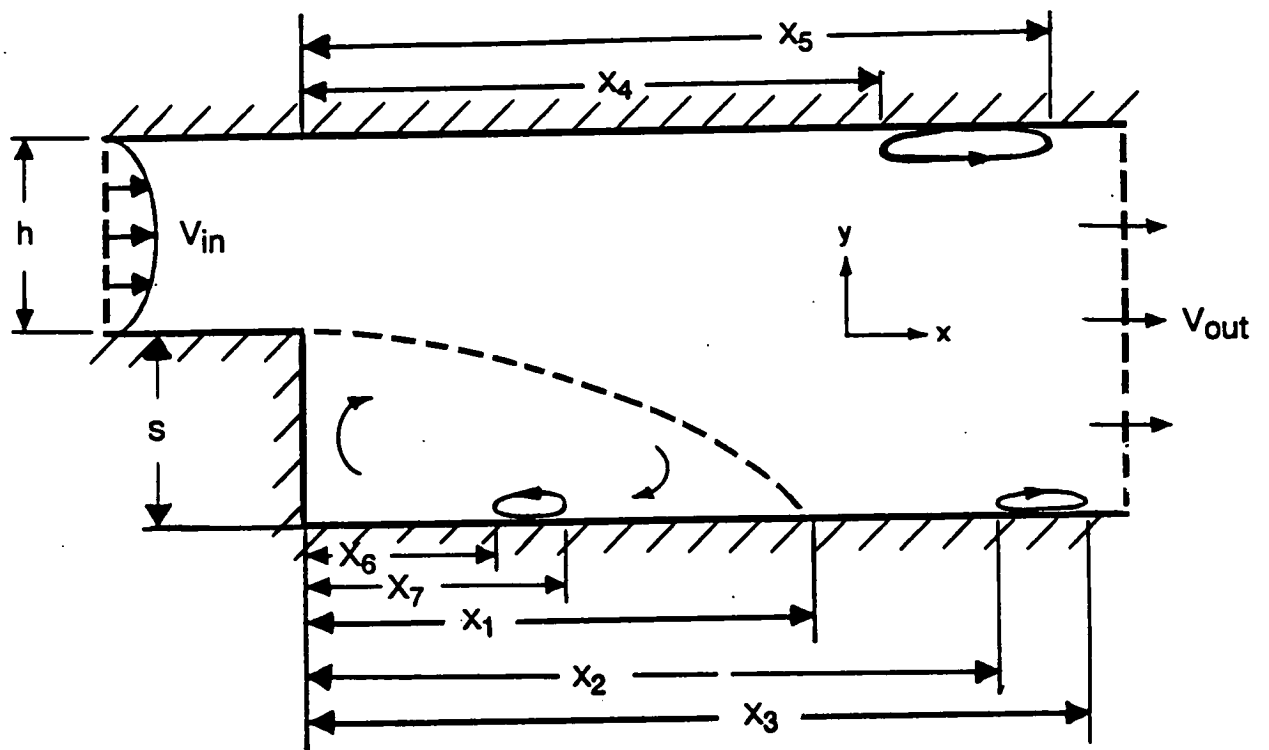


Figure 1. Backward-Facing Step 2-D Flowfield Geometry

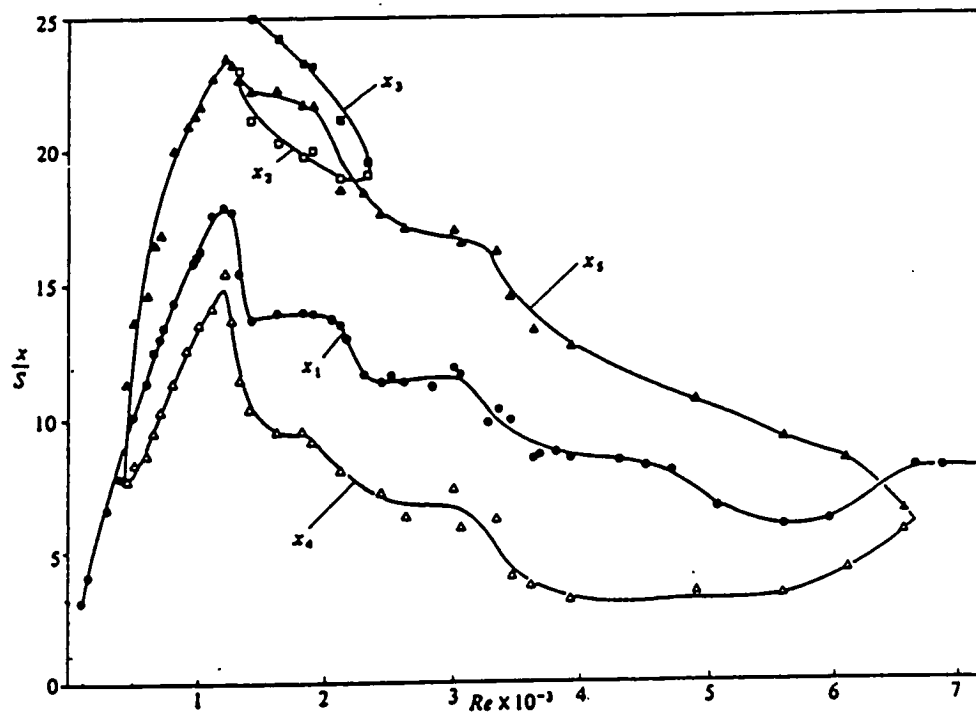


Figure 2. Experimental Results from Reference 149

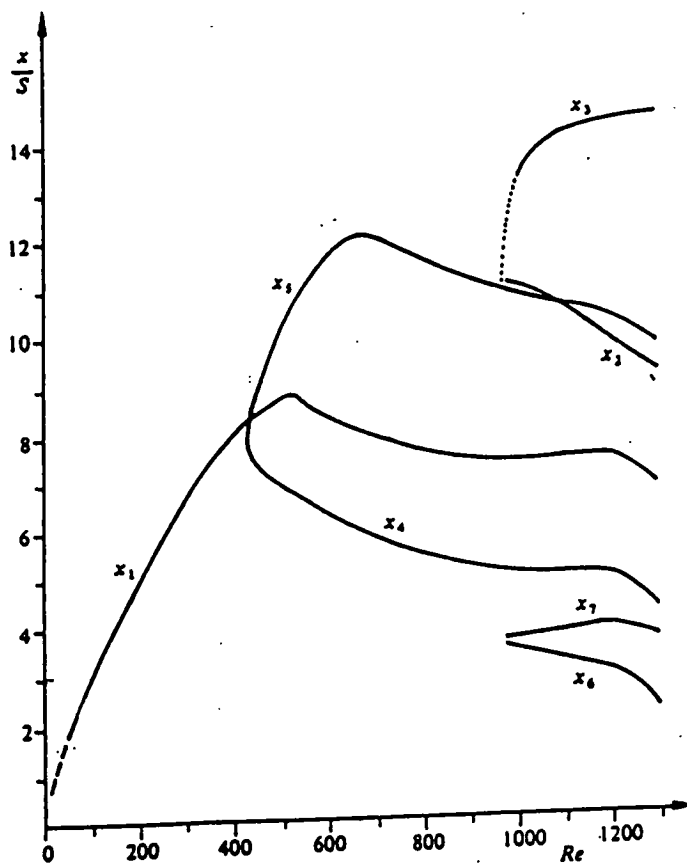


Figure 3. Predicted Results from Reference 149

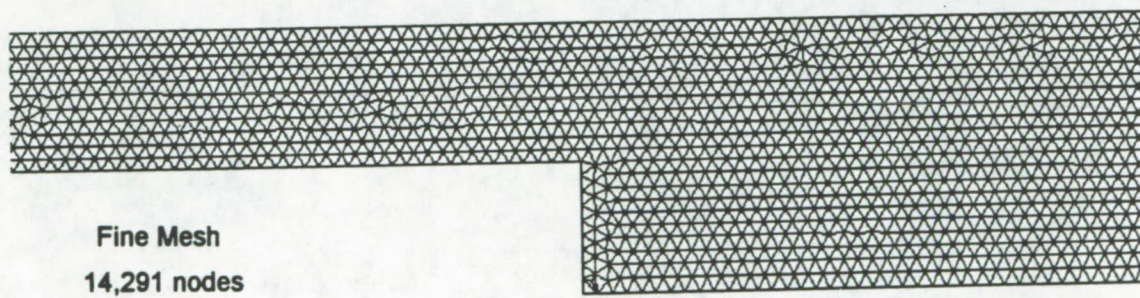
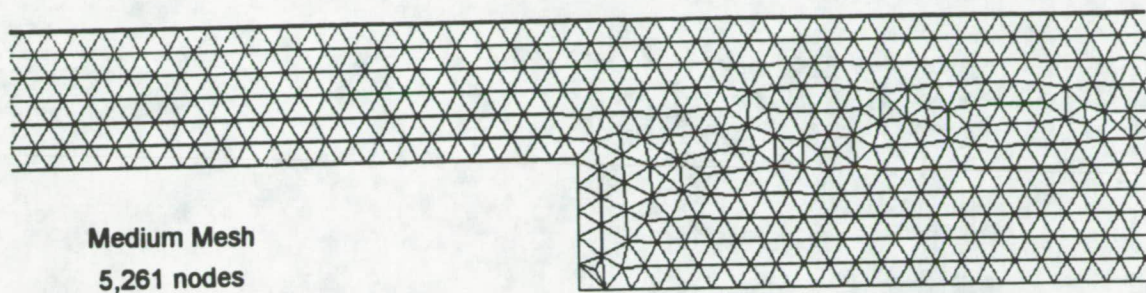
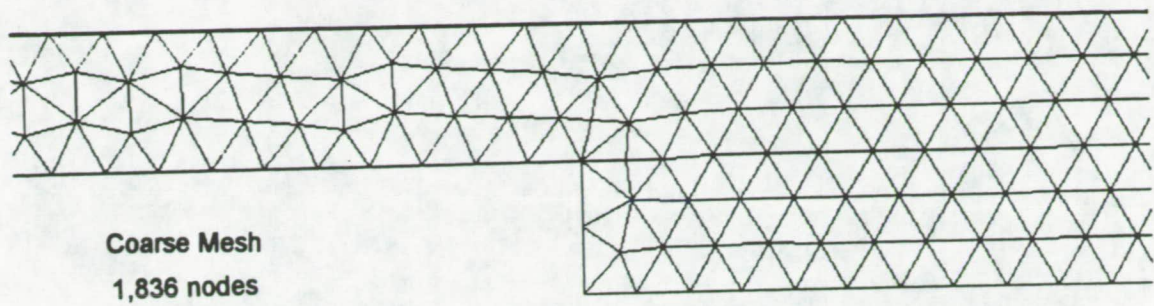


Figure 4. Finite Element Meshes for Backward-Facing Step

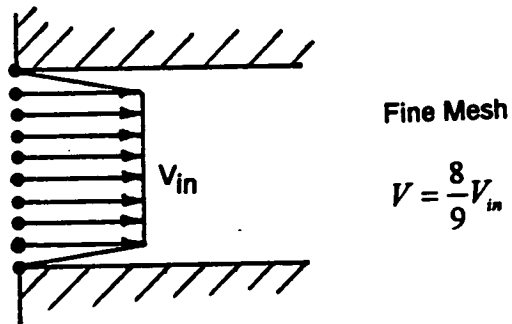
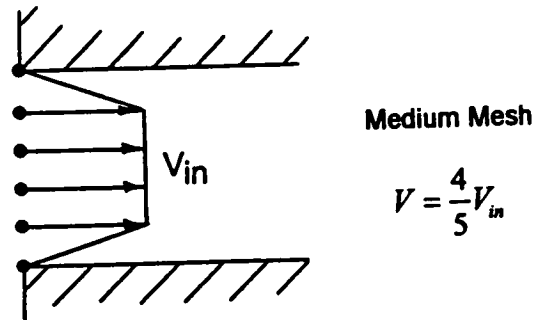
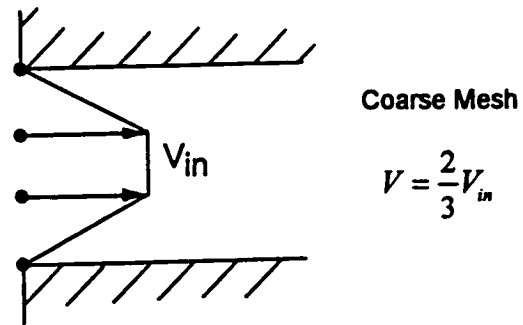


Figure 5. Inlet Average Velocities for Backward-Facing Step

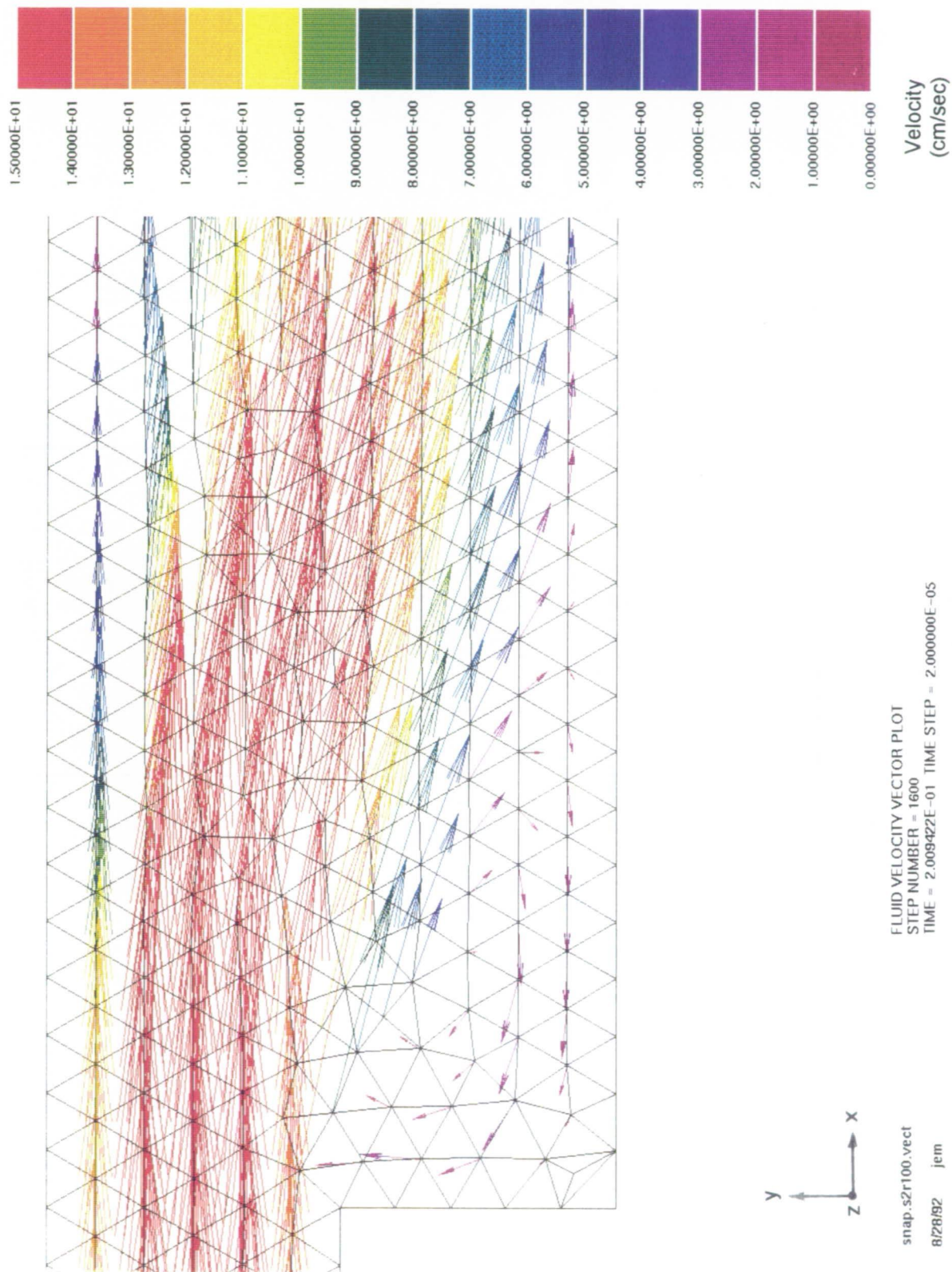


Figure 6. Velocity Vectors for the Medium Mesh, $Re = 100$ Case

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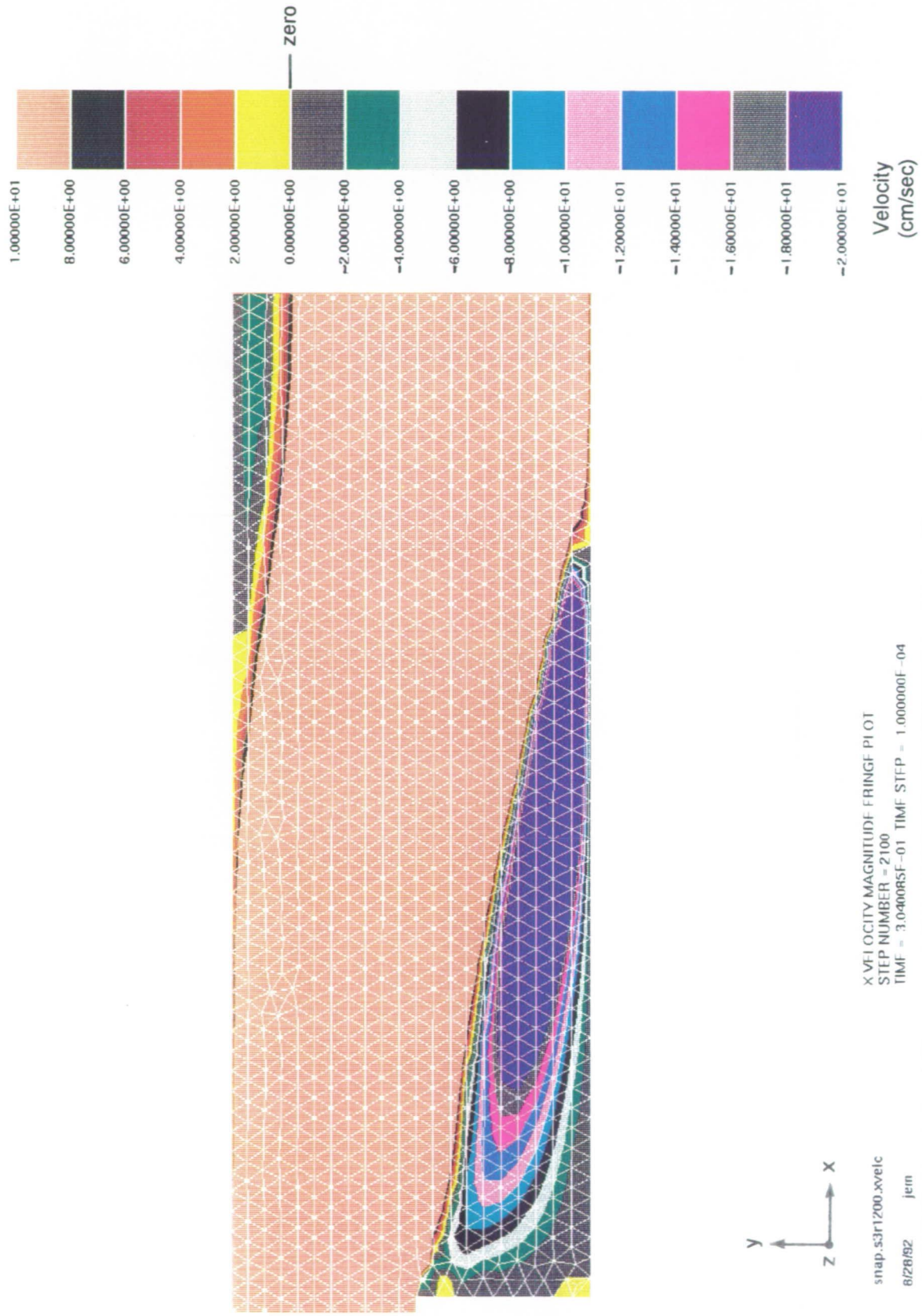


Figure 7. Velocity Contours for the Fine Mesh, $Re = 1,200$ Case

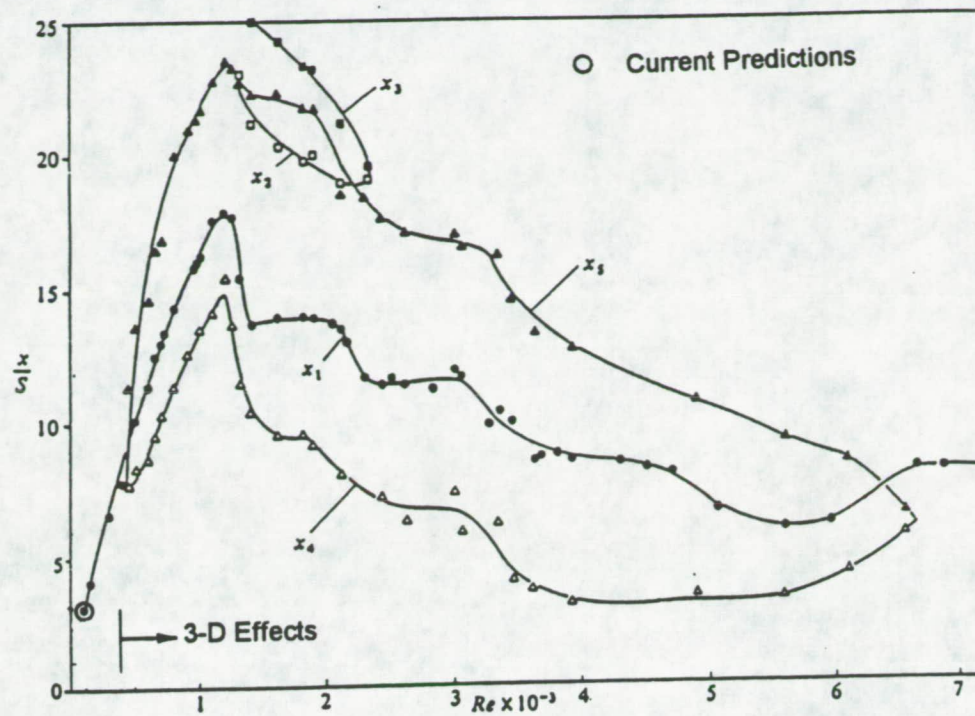


Figure 8. Predicted Results vs. Ref. 149 Experimental Results

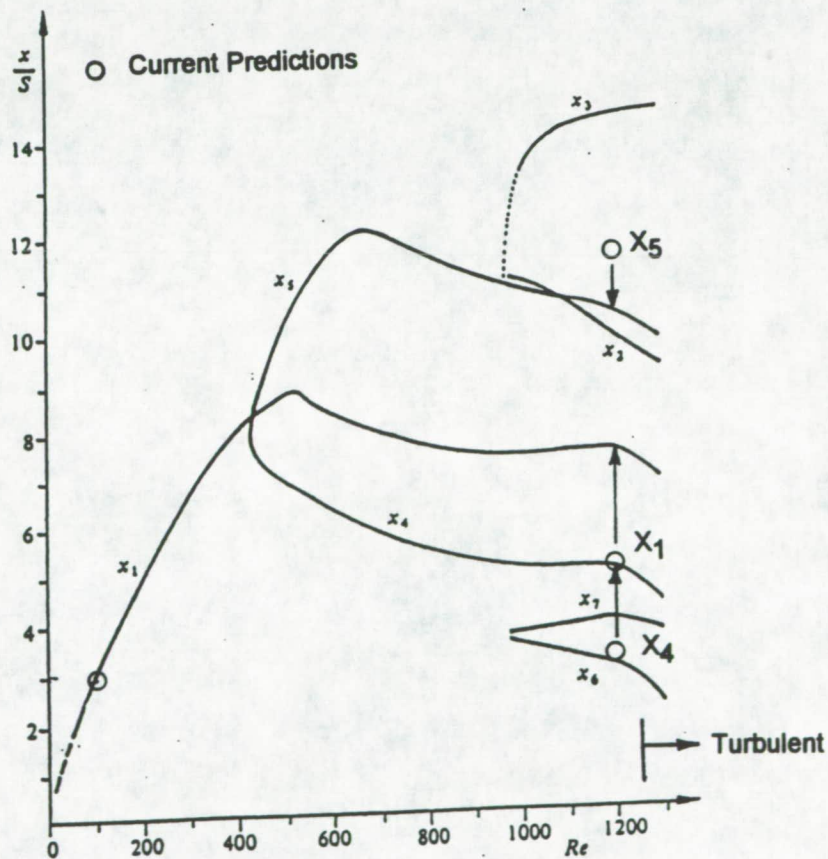


Figure 9. Predicted Results vs. Ref. 149 Predicted Results

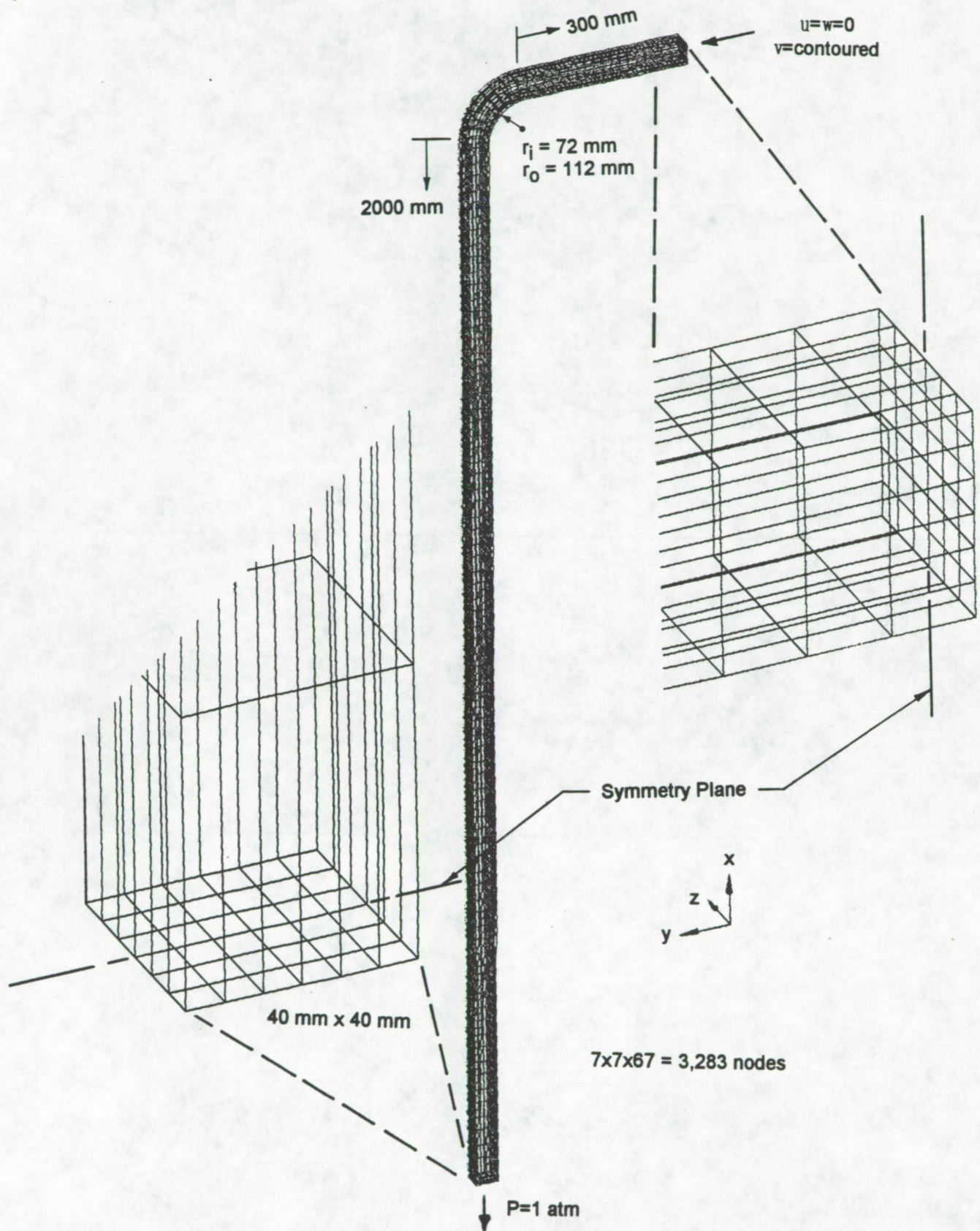


Figure 10. Duct Flow Finite Element Mesh from PATRAN

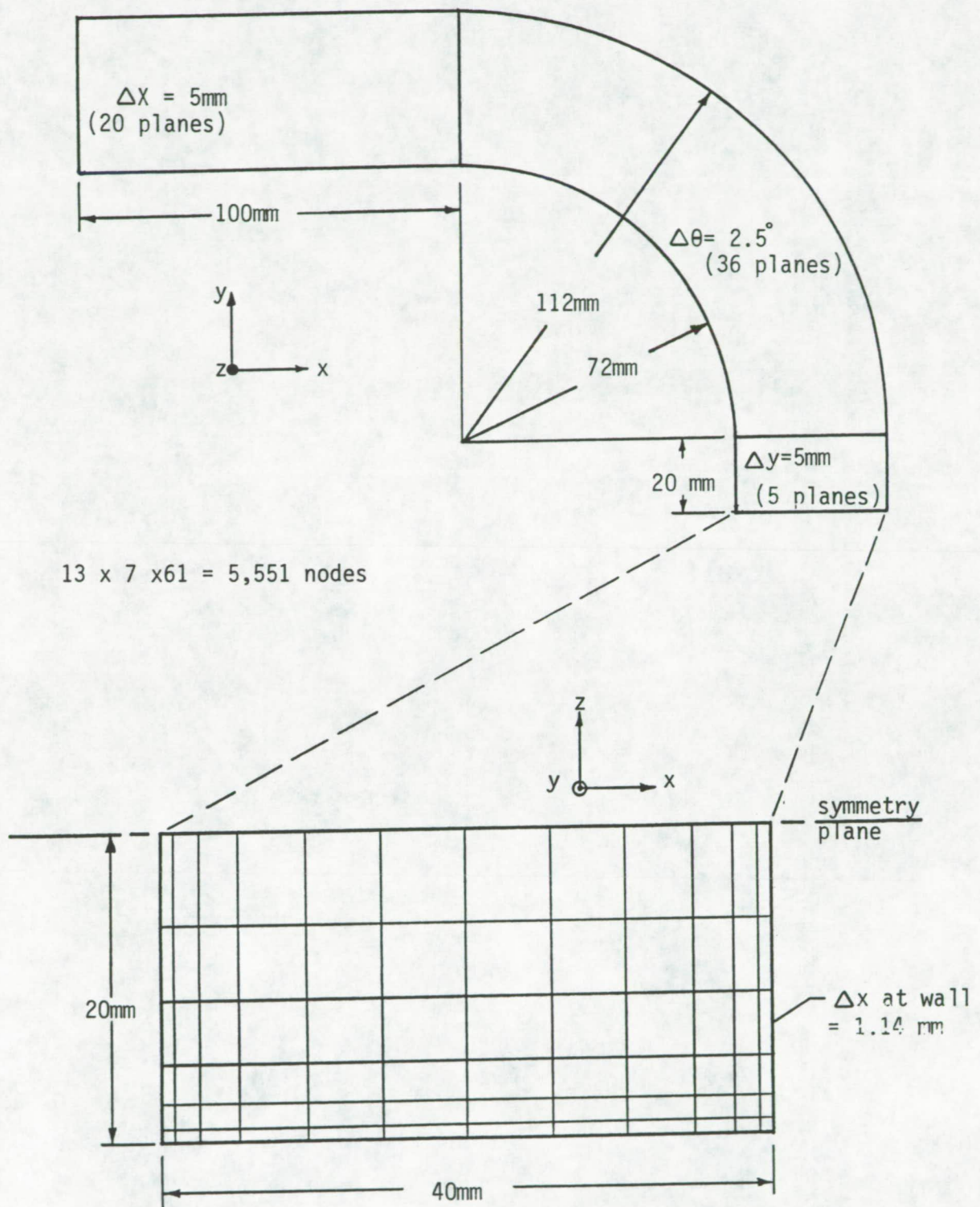
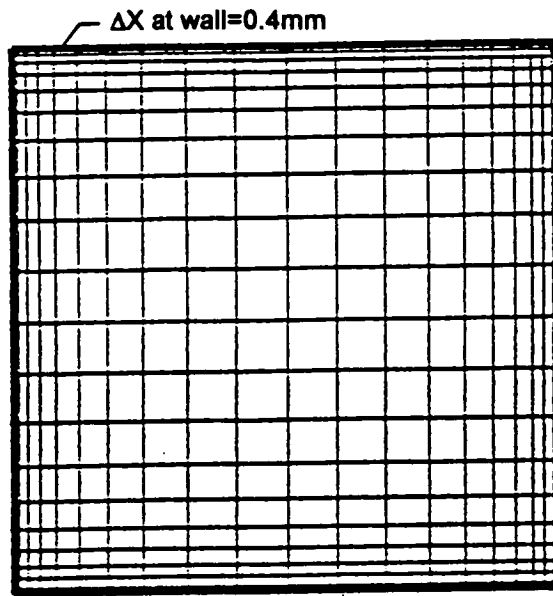
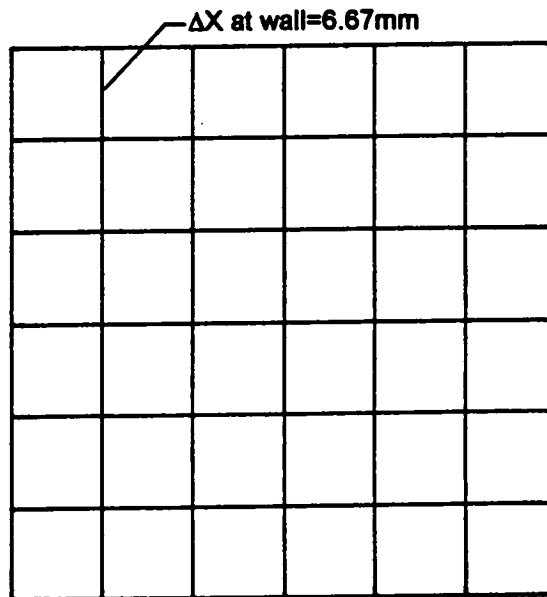


Figure 11. Packed Mesh for Duct Flow

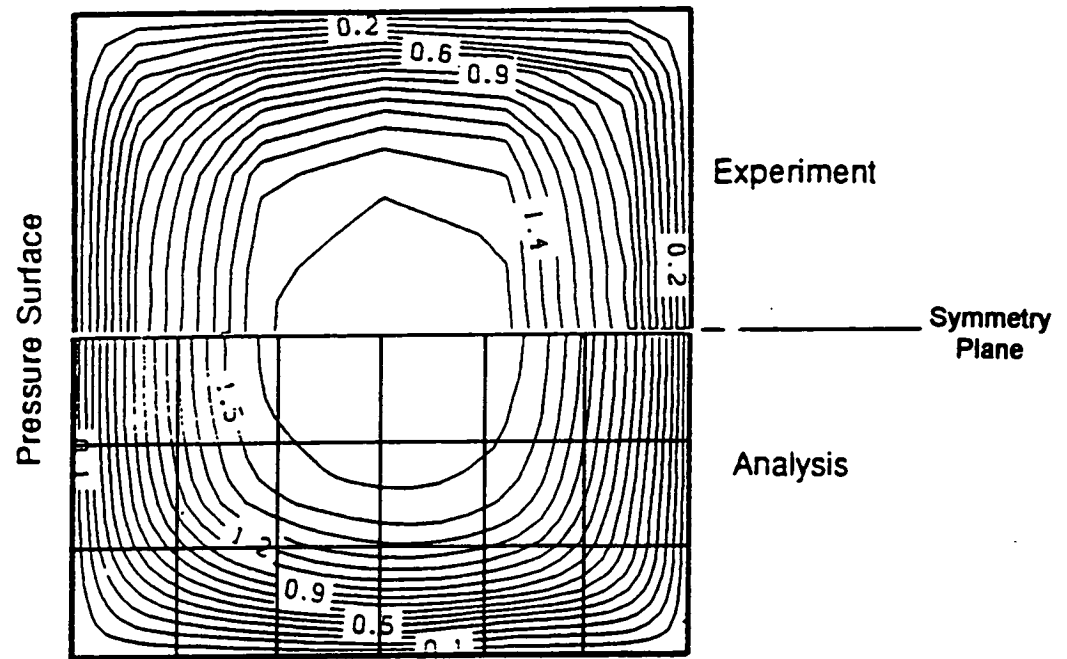


a. 21x21 Mesh from Reference 150

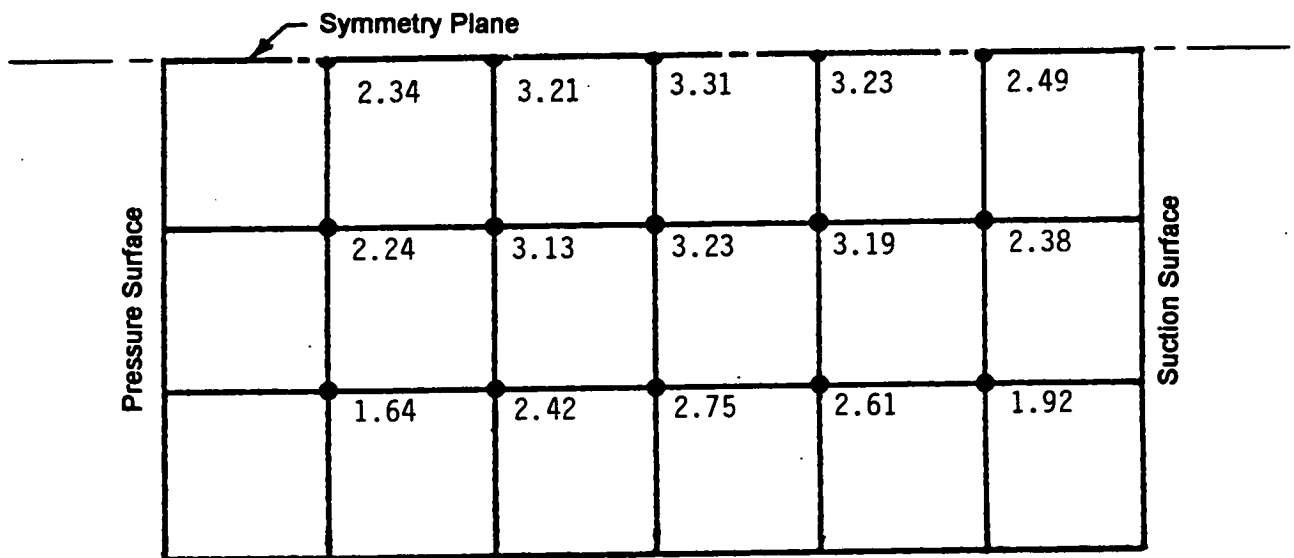


b. 7x7 Mesh of Current Analysis

Figure 12. Comparison of Current Mesh and Ref. 150 Mesh



a. Test Data, Smoothed Contours and 7x7 Overlay



b. Input Velocities in cm/sec from Overlay

Figure 13. Inlet Velocities for Uniform Mesh Duct Flow

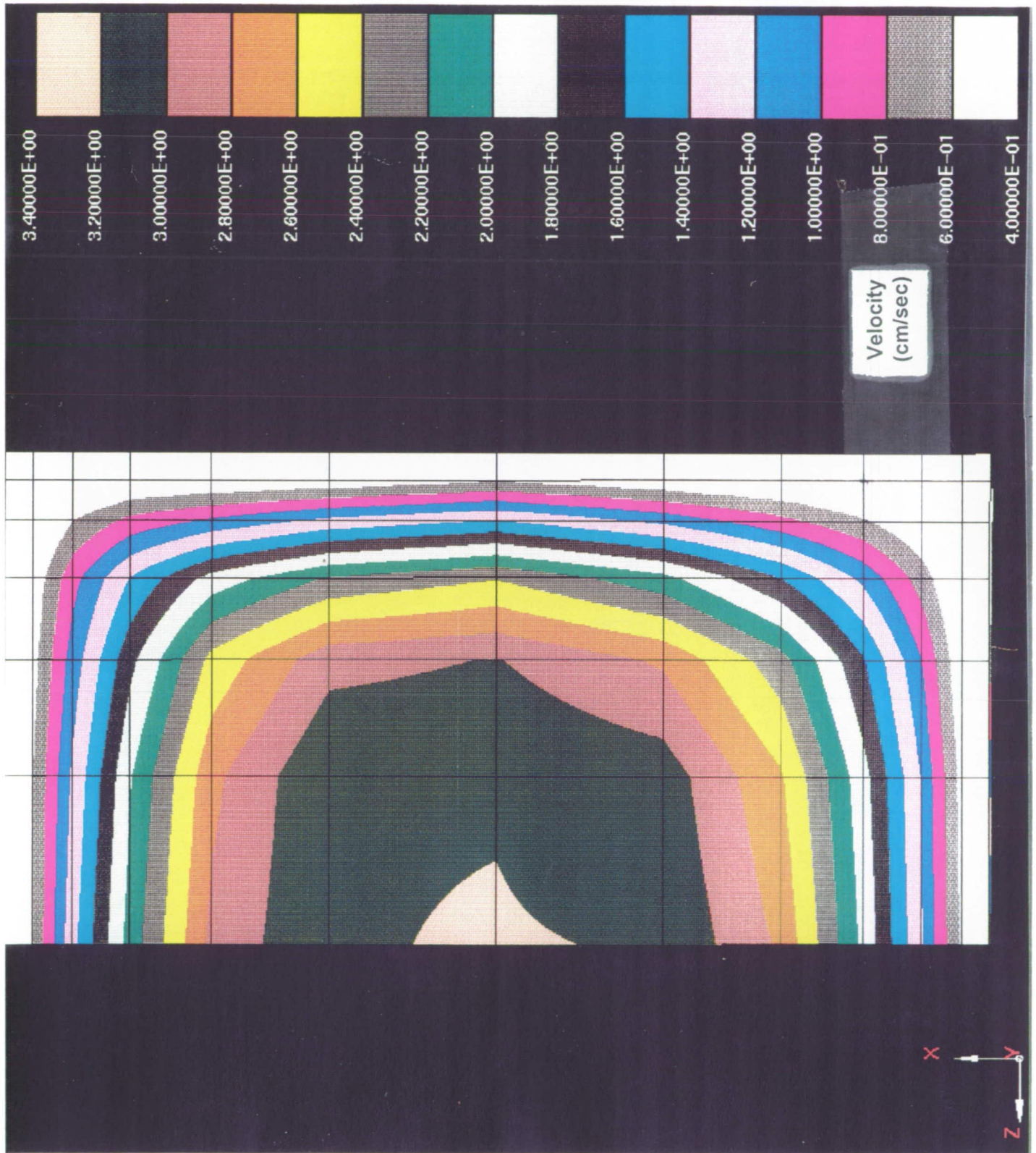
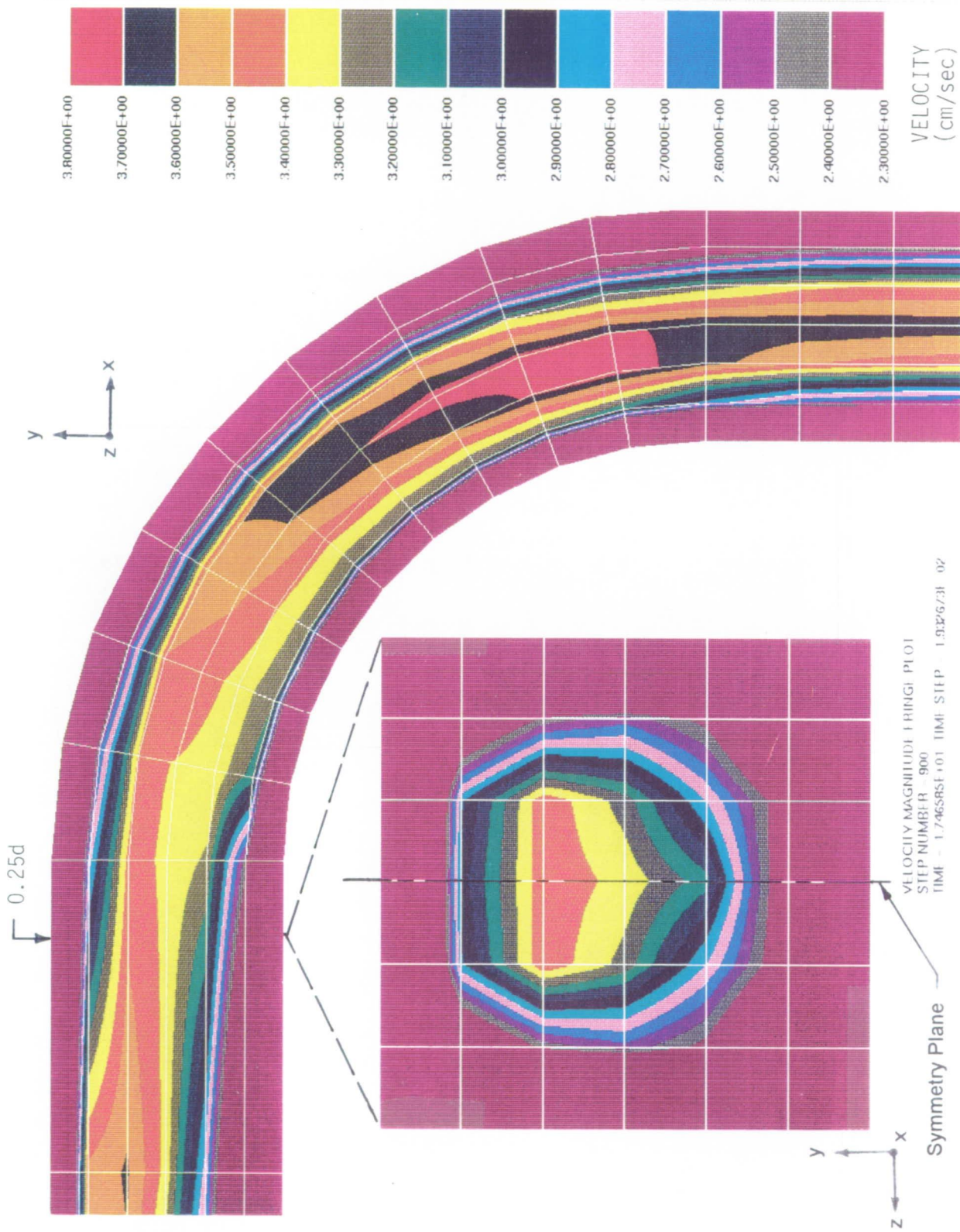


Figure 14. Inlet Velocity Profiles for Packed Mesh Duct Flow



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Figure 15. Velocity Magnitude Contours for Uniform Mesh Duct Flow

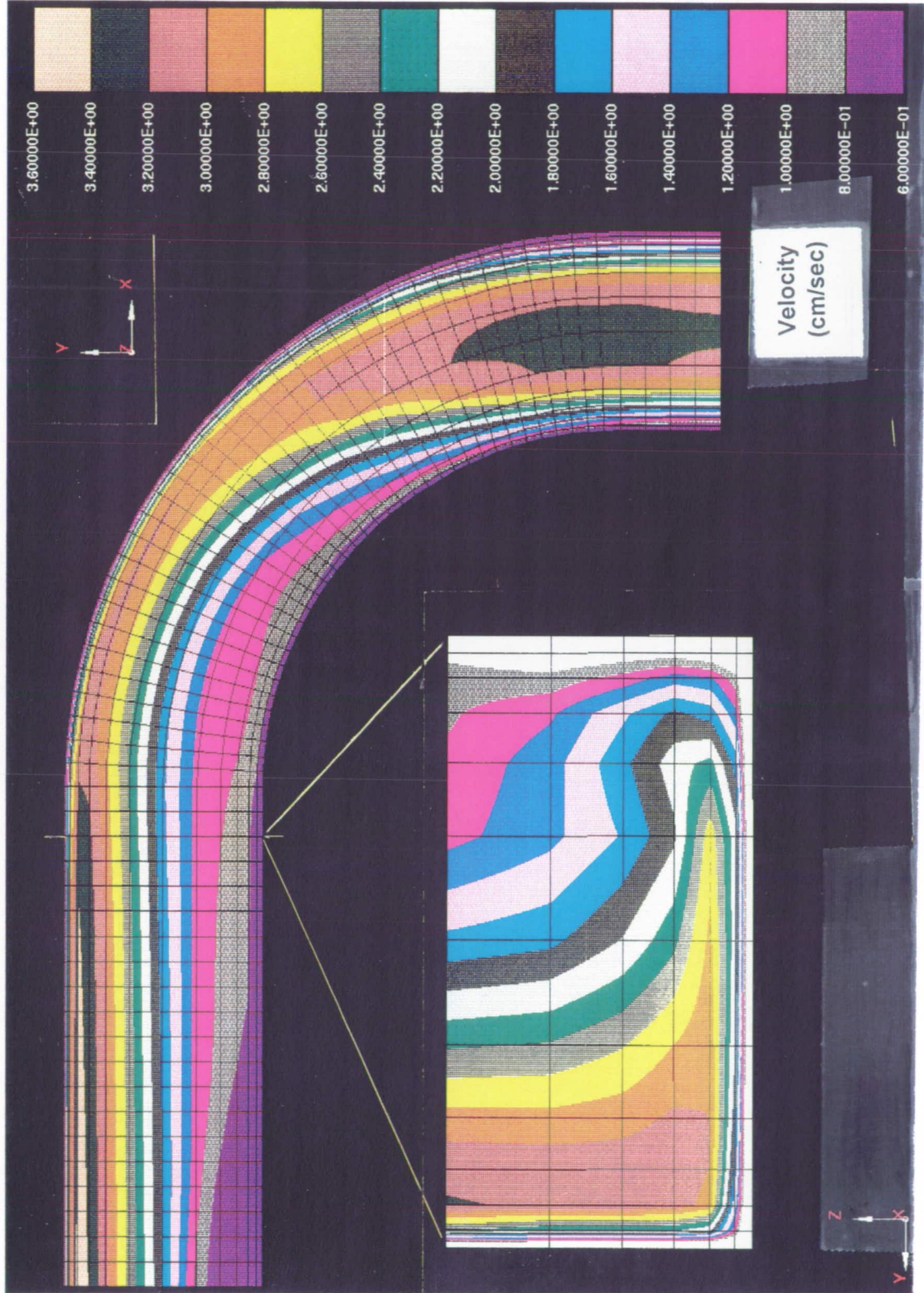


Figure 16. Velocity Magnitude Contours for Packed Mesh Duct Flow

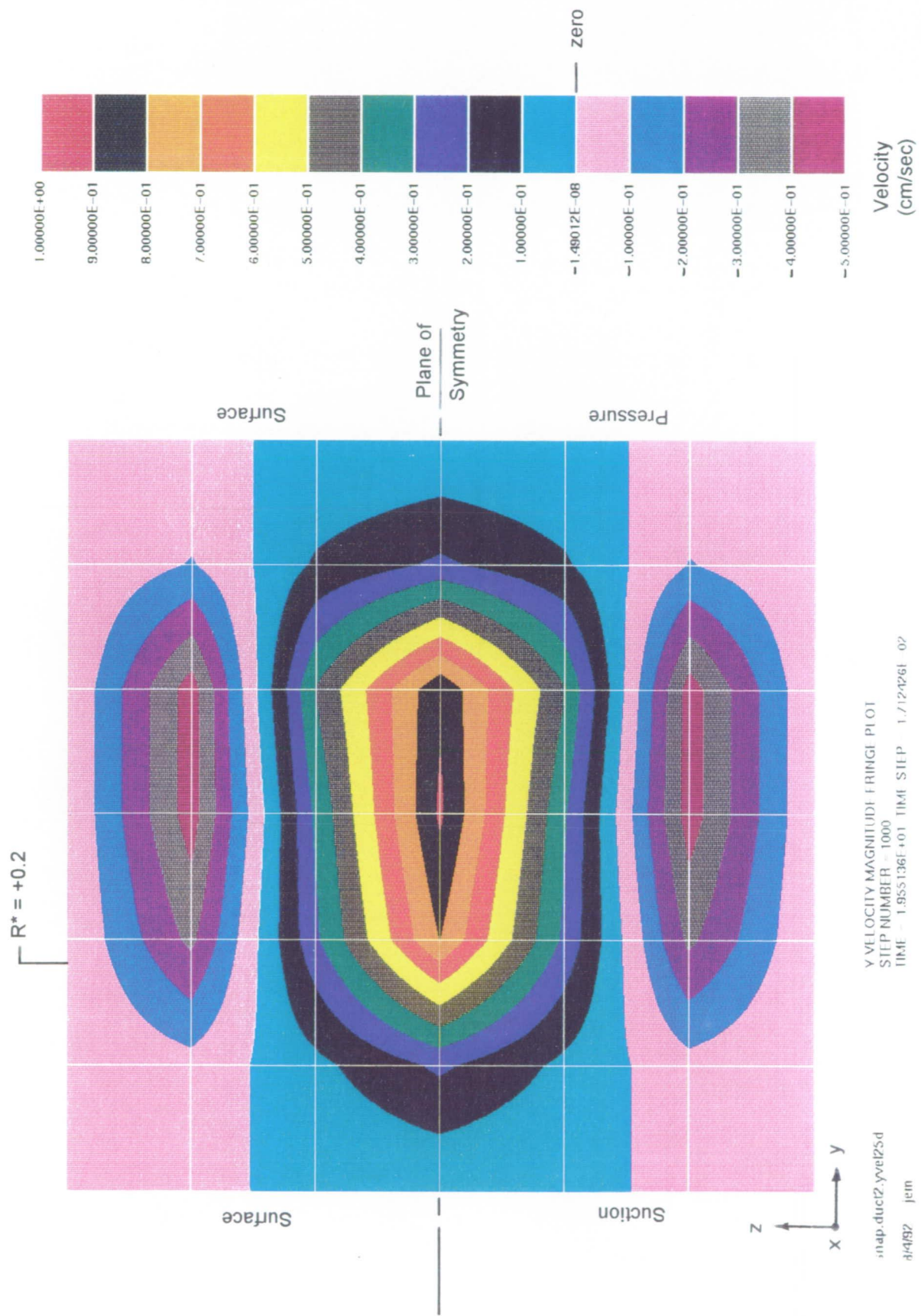


Figure 17. Radial (y-direction) Velocity Contours for Uniform Mesh Duct Flow

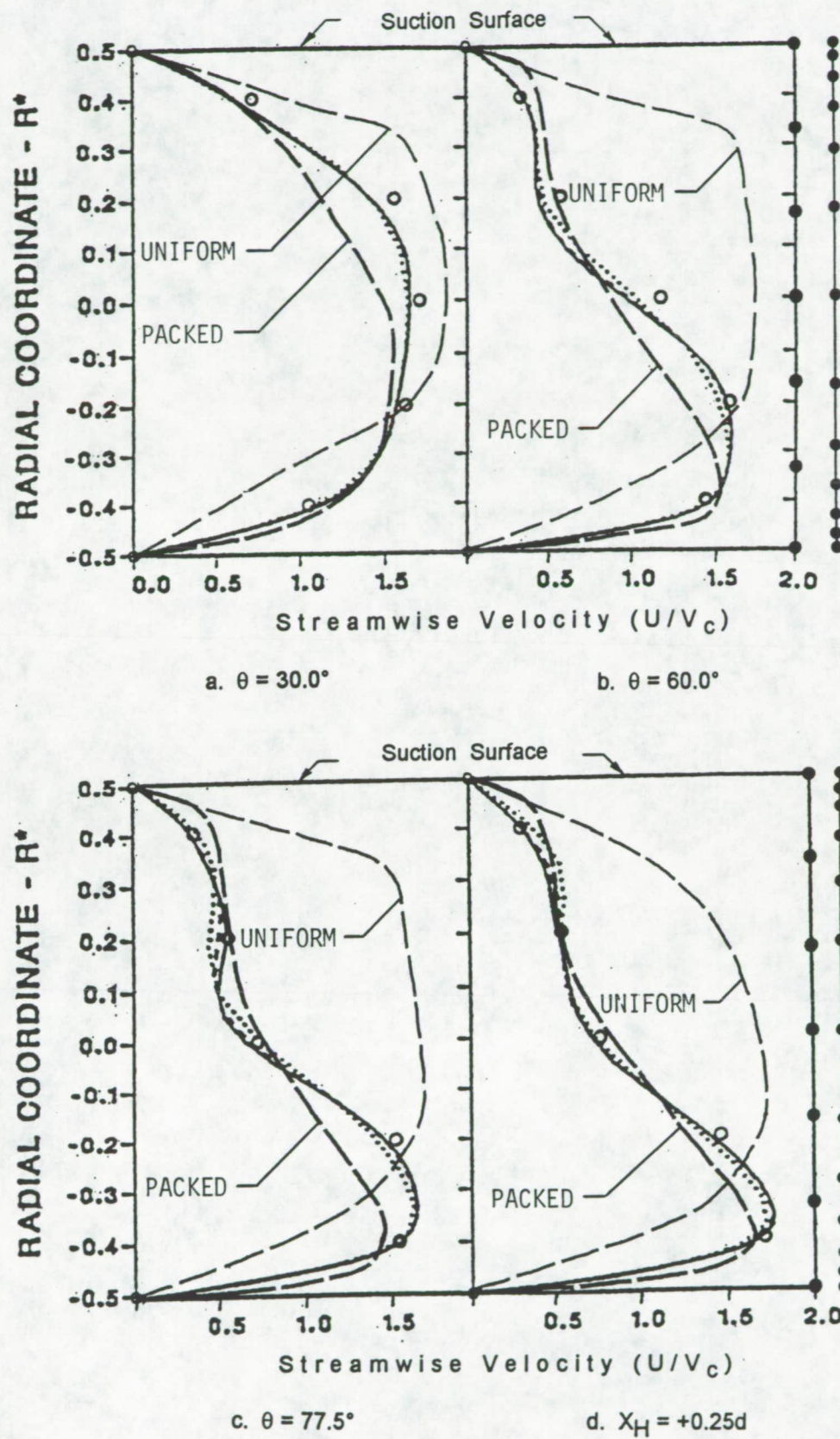
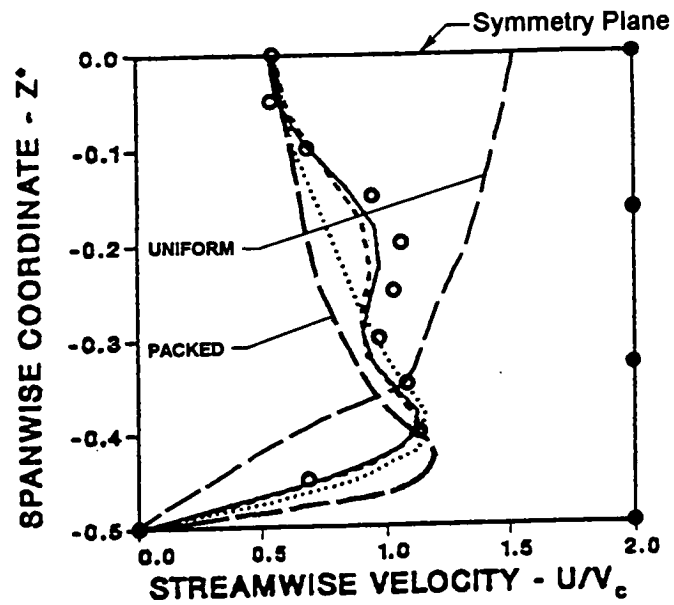
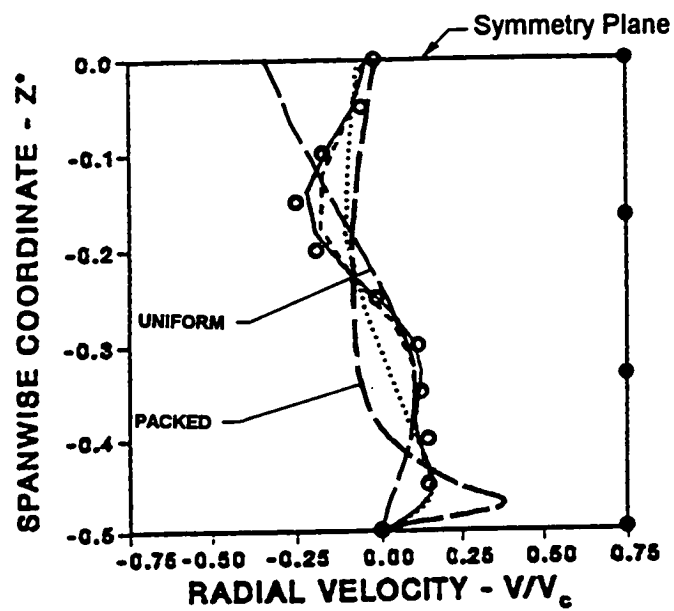


Figure 18. Streamwise Velocity Profiles for Duct Flow



a. Streamwise Velocity Profile at $X_H = +0.25d$ and $R^* = 0.2$



b. Radial Velocity Profile at $X_H = +0.25d$ and $R^* = 0.2$

Figure 19. Velocity Profiles Across the span (z-direction for Duct Flow)

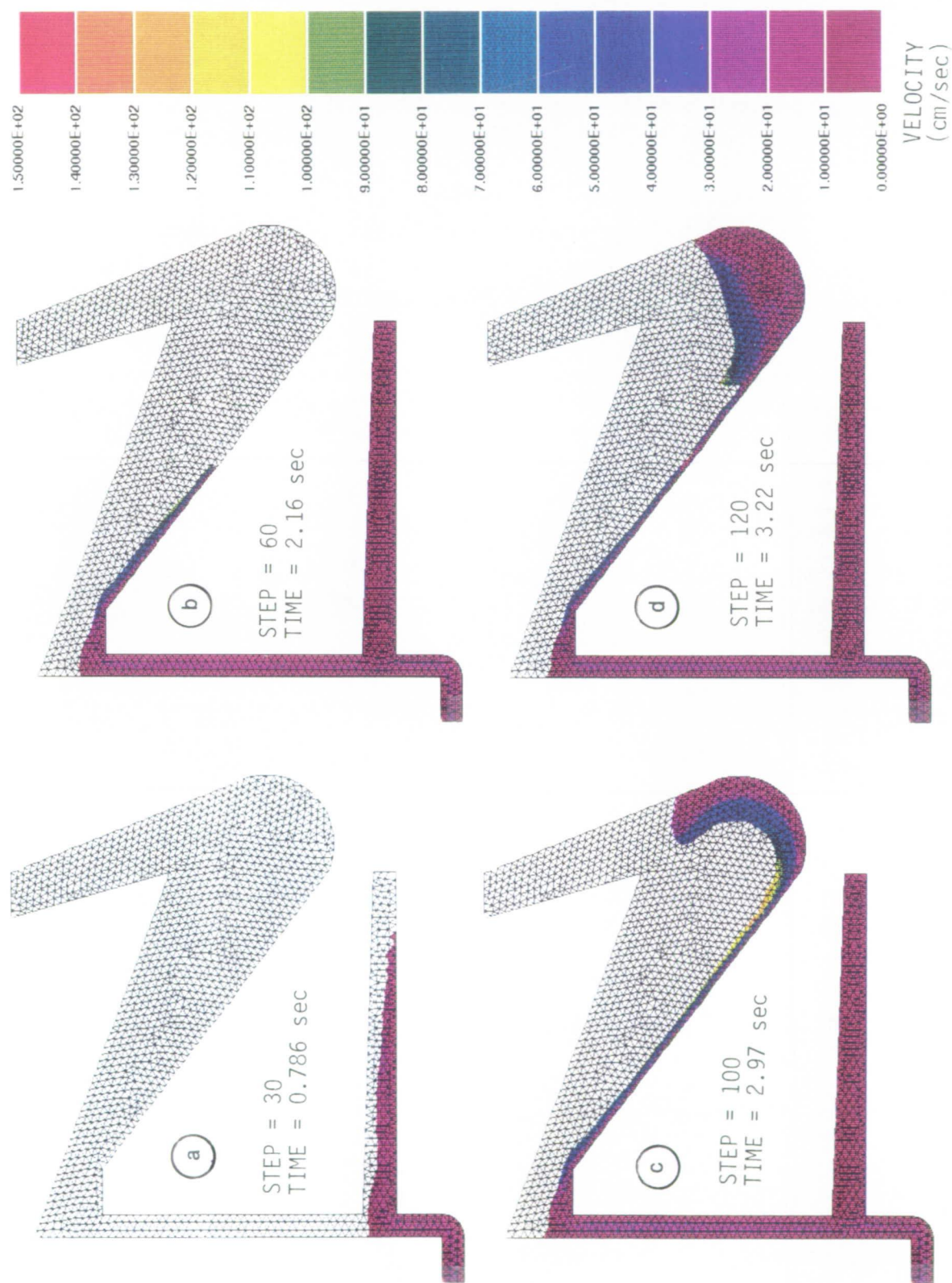


Figure 20. Mold Filling Flowfield Demonstration

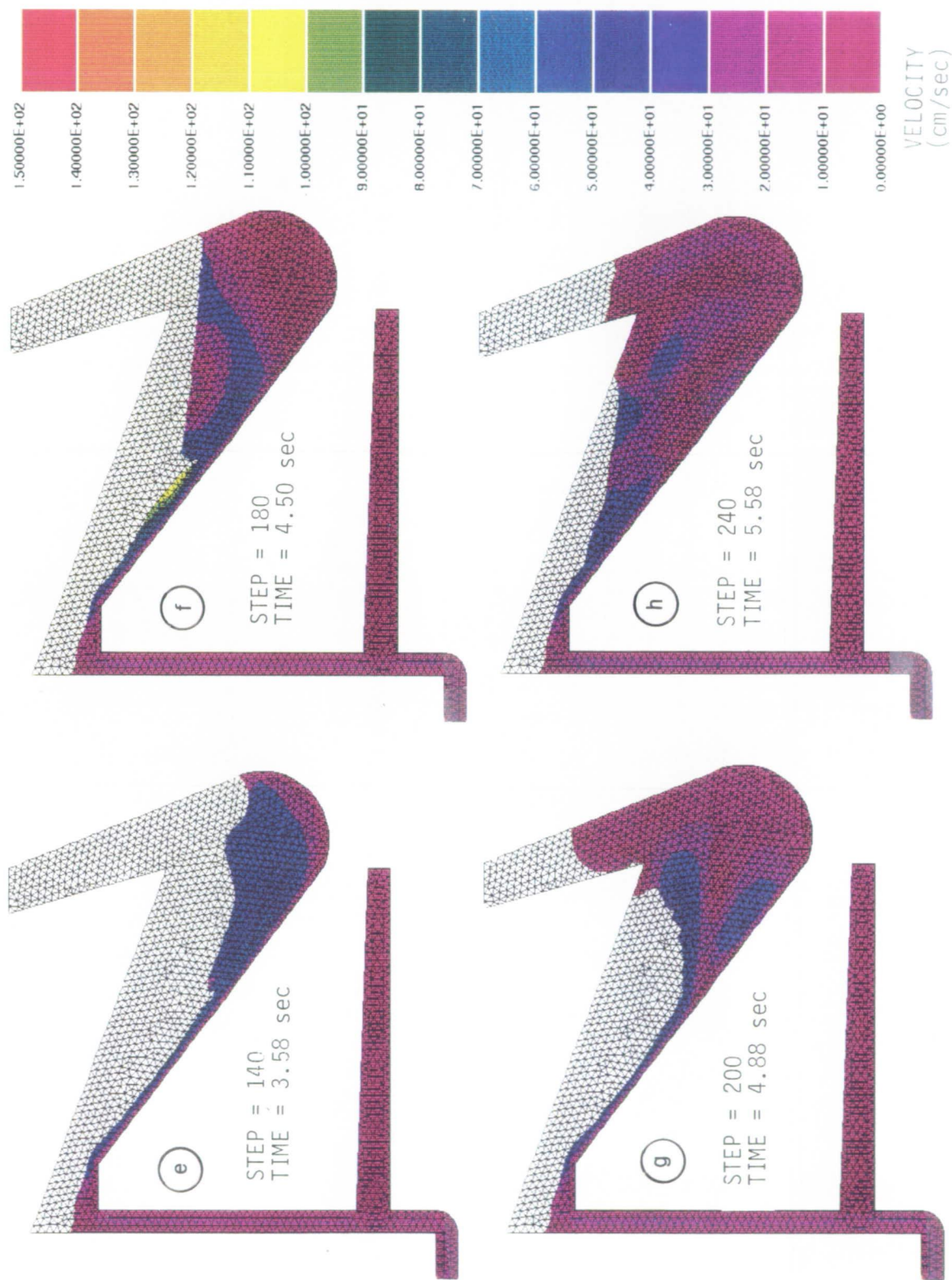


Figure 20. Mold Filling Flowfield Demonstration, continued

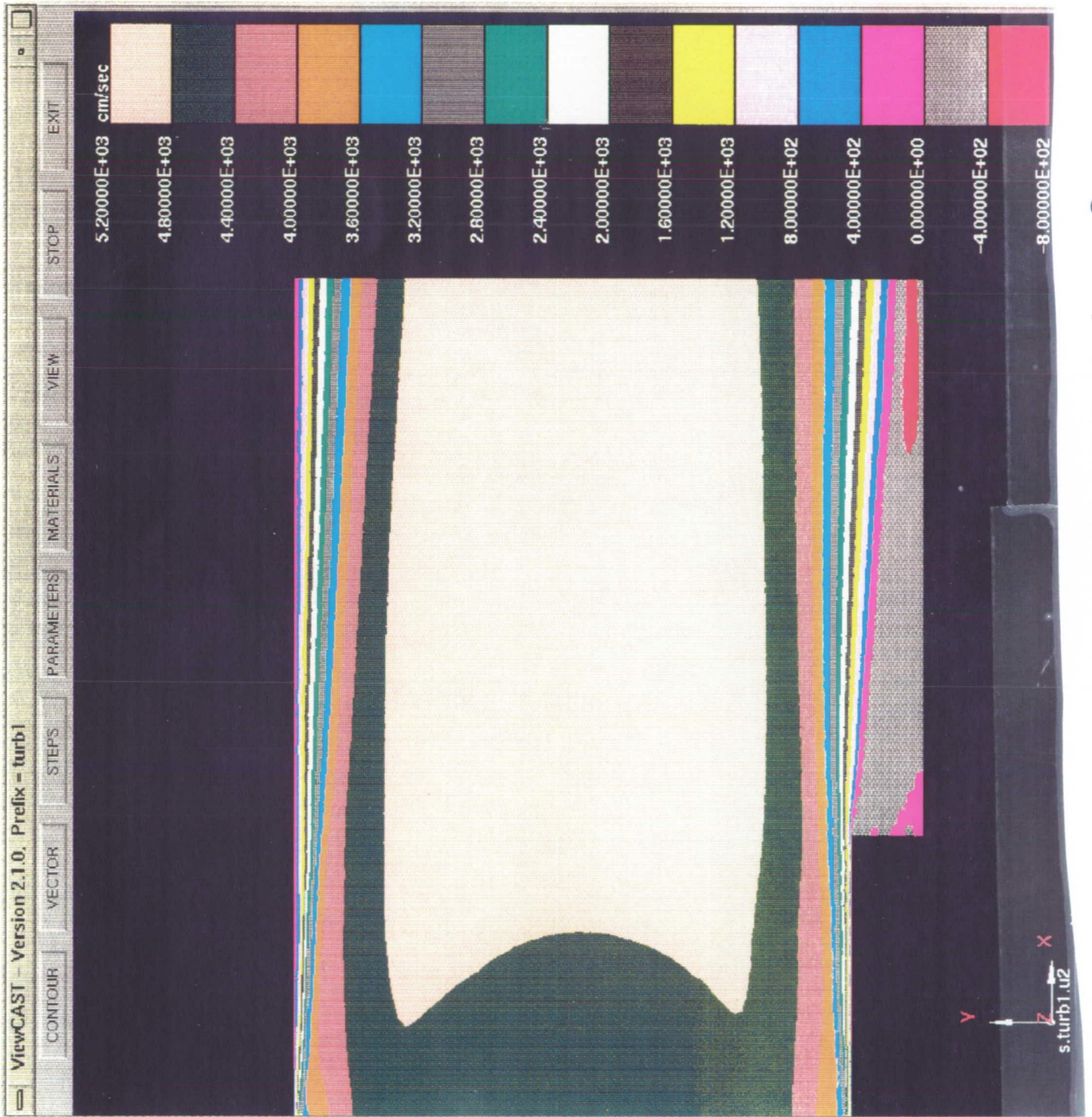


Figure 21. Laminar Solution of Turbulence Case

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