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THE FAVRE - REYNOLDS AVERAGE DISTINCTION AND A CONSISTENT GRADIENT TRANSPORT EXPRESSION FOR THE DISSIPATION

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Abstract ¹

Two equation and higher order closures for compressible turbulence fail to capture the compressible wall layers' log scaling. Accounting for the distinction between Favre and Reynolds averaged variables in the compressible moment equations indicate that turbulent transport expressions obtained using the "variable density approximation" are in error. The error is related to the enstrophy, a *Reynolds averaged* variable appearing in the equation for the *Favre averaged* k ; recognizing this fact an expression for the transport of dissipation consistent with simple mixing length arguments is obtained. Within the (limited) context of a gradient transport hypothesis a rational form for the turbulent transport of the dissipation is found. Modestly better agreement with the well established compressible Van Driest log scaling is found in a $k - \varepsilon$ calculation.

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1. Introduction

Huang *et al.* [1], Wilcox [2] have indicated that current $k-\varepsilon$ turbulence closures do not adequately capture the compressible log layer behavior. Part of the problem with these sorts of turbulence closures appears to result from not adequately recognizing the distinction between Favre and Reynolds averaged quantities - in the Favre averaged equations both Favre and Reynolds averaged variables naturally appear. When mean density gradients are important, not accounting for the distinction contributes to poor results.

In this article a derivation for the modelled dissipation equation for a compressible turbulent flow is described. While not accounting for many of the complex effects of compressibility, discussed in a general sense in Lele [4] or in the context of the fluctuating dilatation *eg.* Ristorcelli [5], the present development, which involves *no modeling assumptions*, shows better agreement with the compressible Van Driest log law scaling. The usual gradient transfer expression for the turbulent transport of the dissipation is found to be written as $\nu_t \nabla(\bar{\rho} \varepsilon_s)$ and not as $\nu_t \bar{\rho} \nabla \varepsilon_s$ as obtained using the so-called “variable density extension”. Huang [3] has communicated that such a form for the turbulent transport will produce better agreement with the Van Driest scaling. Wilcox [2] also comments on the fact that, if $\bar{\rho} \varepsilon$ were the dependent variable, then $k-\varepsilon$ models would have closer agreement with the Van Driest scaling. While there has been speculation, Huang [1] about the properly conserved flow variable, there does not appear to be a clear indication as to what the proper formulation for the dissipation equation should be. This note indicates the appropriate form using a simple, but careful mathematical development.

2. Derivation

The nomenclature is now described. Upper case letters will be used to denote mean quantities and lower case letters fluctuating quantities. Exceptions to this rule are the mean density, $\bar{\rho}$, (ρ has no convenient upper case form) or the mean viscosity $\bar{\mu}$. Quantities with an asterisk denote the total field, mean and fluctuating: $\rho^* = \bar{\rho} + \rho'$ or $u_i^* = U_i + u_i$. Favre velocities will be denoted using the set $[V_i, v_i]$ while Reynolds variables are denoted using the set $[U_i, u_i]$. The dependent variables are decomposed according to $u_i^* = U_i + u_i = V_i + v_i$ where $\langle u_i \rangle = 0$, $\{v_i\} = 0$, and $\rho^* = \bar{\rho} + \rho'$ where $\langle \rho' \rangle = 0$. The averaging operation is indicated using the angle brackets for time means, $\langle v_i v_j \rangle$, and the curly brackets for the density-weighted or Favre mean $\{v_i v_j\}$; the two averages are related by $\bar{\rho} \{v_i v_j\} = \langle \rho^* v_i v_j \rangle$. Without loss of clarity the prime on the fluctuating density is dropped. The portion of the second-order moment equations,

$$\bar{\rho} \frac{D}{Dt} \{v_i v_j\} = P_{ij} + \Pi_{ij} + T_{ijk,k} - \langle u_{j,p} \sigma_{ip}^u \rangle - \langle u_{i,p} \sigma_{jp}^u \rangle, \quad (1)$$

of interest are associated with the viscous dissipation type terms, where $\sigma_{ij}^u = \bar{\mu} [u_{i,j} + u_{j,i} - 2/3 u_{q,q} \delta_{ij}]$.

The quantities P_{ij} , Π_{ij} , $T_{ijk,k}$ are, respectively, the production pressure strain and turbulent transport of the Favre averaged Reynolds stress, $\{v_j v_j\}$. Note that in the Reynolds stress equations the terms arising from the surface forces terms appear naturally in Reynolds, u_i , variables while the problem is posed in Favre, v_i , variables. Repeated application of the product rule for differentiation and the definition of vorticity $\omega_i = \epsilon_{ijk} u_{j,k}$ produces in the equation for the kinetic energy of the turbulence the usual dissipation quantities. Keeping only terms of interest produces

$$\bar{\rho} \frac{D}{Dt} k = -\bar{\rho} \{v_i v_j\} V_{i,j} + T_k - \bar{\mu} \langle \omega_k \omega_k \rangle - \frac{4}{3} \bar{\mu} \langle dd \rangle \quad (2)$$

where $k = \{v_j v_j\}$. The solenoidal dissipation has been rewritten in terms of the enstrophy assuming small scale isotropy. The dilatational dissipation is denoted $\varepsilon_c = \frac{4}{3} \langle dd \rangle$. Our interest is with the portion of the dissipation associated with the vortical modes of the flow. It is conventional to define the solenoidal dissipation in terms of the Reynolds averaged enstrophy: $\bar{\rho} \varepsilon_s = \bar{\mu} \langle \omega_k \omega_k \rangle$. In the “variable density approximation” and high Reynolds log layer limit the dissipation is *assumed* to be described by the high Reynolds number form of the Favre averaged equation

$$\bar{\rho} \frac{D}{Dt} \varepsilon_s - [\bar{\rho} \nu_t \sigma_\varepsilon^{-1} \varepsilon_{s,q}]_{,q} = \bar{\rho} [c_{\varepsilon 1} P_k - c_{\varepsilon 2} \varepsilon_s] \varepsilon_s / k. \quad (3)$$

where $P_k = -\{v_i v_j\} S_{ij}$. $S_{ij} = \frac{1}{2} [V_{i,j} + V_{j,i} - \frac{2}{3} D \delta_{ij}]$ in which $D = U_{p,p}$ is the mean dilatation which is small in the log layer portion of this flow. Note the location of the mean density in the turbulent transport term. The enstrophy’s replacement by what is treated as a Favre averaged variable, ε_s , must be done carefully. The conservation equation for the enstrophy, the primitive Reynolds averaged variable appearing in the Reynolds stress equations, is, to lowest order,

$$\langle \omega^2 \rangle_{,t} + U_q \langle \omega^2 \rangle_{,q} + \langle u_q \omega^2 \rangle_{,q} = -\frac{4}{3} \langle \omega^2 \rangle D + 2 \langle \omega_i s_{ij} \omega_j \rangle - \bar{\mu} \langle \omega_{i,k} \omega_{i,k} \rangle \quad (4)$$

The last two terms in the above equations are modeled by the right hands side of equation 3. The gradient transport hypothesis for the enstrophy transport produces $\langle u_q \omega^2 \rangle = -\nu_t \langle \omega^2 \rangle_{,q}$, where $\nu_t = c_\mu k^2 / \varepsilon$. Replacing the enstrophy with the dissipation using $\bar{\rho} \varepsilon_s = \bar{\mu} \langle \omega_k \omega_k \rangle$ produces

$$\bar{\rho} \frac{D}{Dt} \varepsilon - [\nu_t \sigma_\varepsilon^{-1} (\bar{\rho} \varepsilon)_{,q}]_{,q} = -\frac{1}{3} \bar{\rho} \varepsilon D + \bar{\rho} [c_{\varepsilon 1} P_k - c_{\varepsilon 2} \varepsilon_s] \varepsilon_s / k. \quad (5)$$

The substantial derivative is along the Favre mean streamline. Note the location of the mean density in the turbulent transport term. In general the appearance of the mean density inside the first derivative will only be of importance when the mean density gradients are large, $\frac{\nabla \bar{\rho}}{\bar{\rho}} \sim \frac{\nabla \varepsilon}{\varepsilon}$. For weak density gradients the variable density assumption and the present derivation will give the same computational result. Calculations, to be shown shortly, were performed with a standard $k - \varepsilon$ model with the usual gradient transport models for transport. The following values for the empirical constants were used: $c_\mu = 0.09$, $c_{\varepsilon 1} = 1.44$, $c_{\varepsilon 2} = 1.92$, $\sigma_k = 1.0$, $\sigma_\varepsilon = 1.17$.

The better performance of the $k - \omega$ type of turbulence models comes from, in part, the fact that $\omega = \varepsilon/k$ is not a specific (*per unit mass*) variable and the Favre form of the convection operator is the correct expression for its turbulent transport.

3. Conclusion

One can follow Huang *et al.*'s [1] mean density gradient contribution analysis to assess the impact of these new forms of the transport terms in the k and ε equations. The coefficients multiplying the density gradient terms in their equation (20) are notably smaller using the present formulation. Wilcox [2] has conducted a similar analysis in which he arrives at a similar conclusion. The implication of both these analyses is that the deviation from the law of the wall will be smaller with these changes to the transport terms. This is substantiated in Figure 1 in which a Mach 8 boundary layer calculation using a $k - \varepsilon$ turbulence model is shown. The top line corresponds to “variable density approximation”; the lower line(s) which are virtually indistinguishable correspond to the empirical compressible log law and an incompressible calculation. The middle lines are with the new form of the transport in the dissipation equation given above: it shows a modest improvement. The upper middle line is without the bulk dilatation term. The point of this article is to present a rational form for the dissipation equation without adding *ad hoc* corrections. The compressible moment equations are, of course, complicated by several additional issues: the effects of the fluctuating dilatation and correlations involving the fluid property fluctuations, for example. These are the subject of current research and are expected to make additional contributions towards a better agreement with the well established Van Driest scaling.

To summarize: by careful accounting for the distinction between Favre and Reynolds averaged variables a different expression for the turbulent transport of the dissipation has been derived. The expression is consistent with mixing length arguments adequate for wall layers. Calculations for the Mach 8 boundary layer show that following a rational procedure produces a closer agreement with the log law of the wall using the conventional models for the other terms in the modeled dissipation equation.

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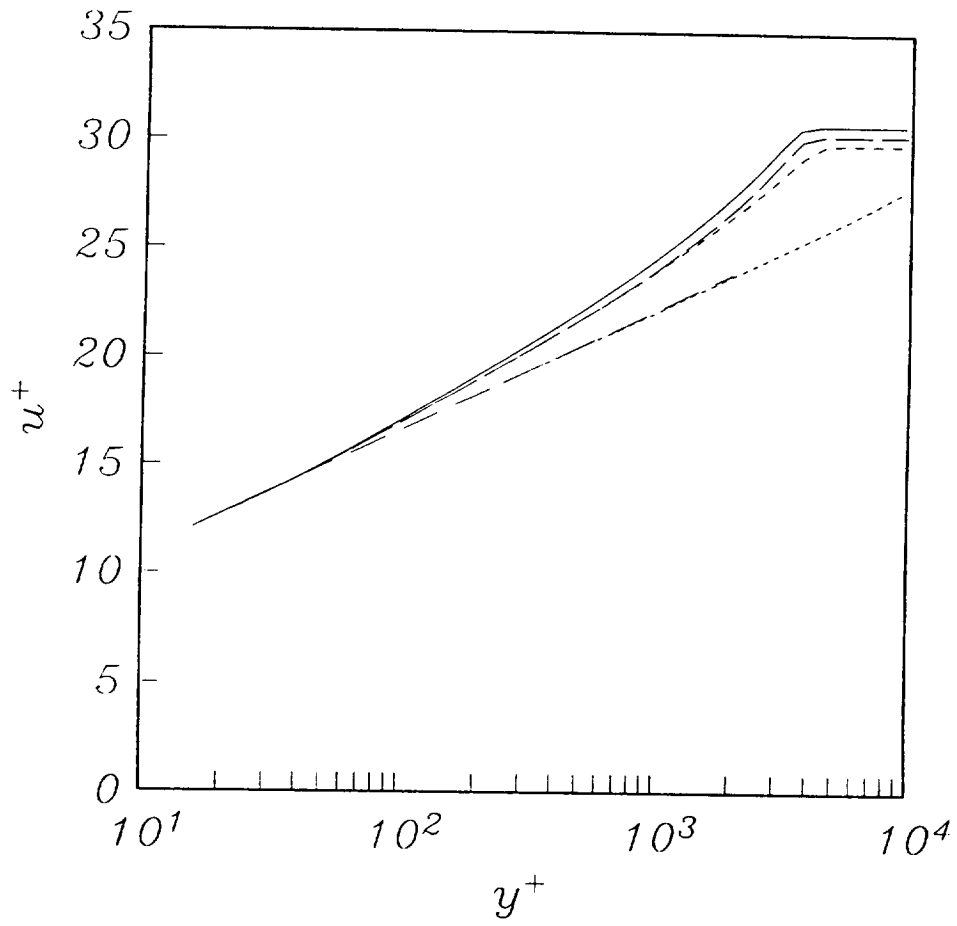


Figure 1. The mean profile for the Mach 8 boundary layer: upper line - variable density approximation; lower lines - the log law and the incompressible calculation; middle lines - the present theory with and without (upper) bulk dilatation.

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