

Global Cost and Weight Evaluation of Fuselage Side Panel Design Concepts

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FOREWORD

This contractor report was prepared by the Boeing Commercial Airplane Group, Renton Washington under contracts NAS1-18889 and NAS1-20013 (Task 2). It documents the global evaluation of fuselage side panel design concepts performed between September 1992 and February 1994. The contracts funding this effort were NAS1-18889 and NAS1-19349 (Task 3), sponsored by the National Aeronautics and Space Administration, Langley Research Center (NASA-LaRC) as part of the Advanced Composite Technology (ACT) program. William T. Freeman, John G. Davis, Mark J. Stuart, and James Starnes were the NASA-LaRC ACT Boeing Contract Monitors.

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1.0 SUMMARY

This report documents preliminary design trades conducted for a subsonic wide body commercial aircraft fuselage side panel section utilizing composite materials. This work was performed during the time period from September 1992 to February 1994 under NASA contracts NAS1-18889 (Advanced Technology Composite Aircraft Structures, ATCAS) and NAS1-19349 (Task 3, Pathfinder Shell Design).

Included in this effort were (a) development of two complete design concepts, (b) generation of thorough cost and weight estimates for both designs, (c) identification of technical issues and potential design enhancements, and (d) selection of a single design to be further developed during subsequent structural and manufacturing process optimization. The first design concept featured an open-section stringer stiffened skin configuration while the second was based on honeycomb core sandwich construction. To provide a reference for the composite designs, a comparable metallic fuselage structure was also developed.

The trade study cost and weight results were generated from comprehensive assessment of each structural component comprising the fuselage side panel section from detail fabrication through airplane final assembly. ACT program groundrules were followed to assure consistency with past work performed. Results were obtained in three phases: (1) for the baseline designs, (2) after global optimization of the designs, and (3) the results anticipated after detailed design optimization. The estimated cost and weight for each concept considered after this final step are provided in Table 1-1 relative to the metallic baseline.

	Cost (\$M, per 300 shipsets)	Weight (lbs, per shipset)
Composite Skin-Stringer	0.77	0.77
Composite Sandwich	0.59	0.71
Metallic Baseline	1.00	1.00

Table 1-1. Side Panel Global Cost and Weight Evaluation Results

In addition to the various cost and weight estimates, a critical assessment of both designs was performed to determine the risk associated with each concept, that is the relative probability of achieving the cost and weight projections. Based upon this assessment, the sandwich design was selected as having the greatest potential to meet the ACT program goals of greater than 20 percent cost savings and more than 30 percent weight savings relative to comparable metallic aircraft structure. Finally, as the first step towards side panel detailed design optimization, seven critical technical issues were identified as listed in Table 1-2. Successful resolution of these issues will be the prime focus of the remainder of Boeing's ACT Phase B program, scheduled for completion in September 1995.

- Keel & Side Panel Detailed Design Development
- Keel Manufacturing & Structural Scaleup
- Side Manufacturing & Structural Scaleup
- Sandwich Panel Repair
- Liquid Ingression Analysis & Test Evaluation
- Cost-Effective, Light-Weight Sandwich Closeouts
- Sandwich Damage Tolerance & Bonded Frame Integrity

Table 1-2. Critical Technical Issues to be Resolved for Successful Demonstration of Composite Honeycomb Sandwich Fuselage Side Panel Structure

2.0 INTRODUCTION

Boeing's Advanced Technology Composite Aircraft Structure (ATCAS) program, initiated in May 1989 under NASA contract NAS1-18889, is an integral part of the NASA Advanced Composite Technology (ACT) program. Reducing the cost and weight of conventional metallic commercial aircraft primary structures by more than 20 percent and 30 percent, respectively, is the goal of the ACT program. (These reduction goals are for a resized aircraft which may achieve as much as 10 percent weight reduction due to synergism of the individual structural components.) Innovative structural concepts utilizing advanced composite materials are being pursued to realize these efficiencies. The specific objective of the ATCAS program is to develop and demonstrate the integrated technology base required to make cost- and weight-effective use of advanced composite materials in pressurized fuselage primary structure.

The ATCAS team has adopted a three-step process to meet the program objective. This approach, applied in an environment integrating all pertinent functional disciplines, is depicted schematically in Figure 2-1 and is fully described in Reference 1. The first step consists of selection of a baseline "design family" for each area of the fuselage: crown, keel, and side panel regions. (As illustrated in Appendix A, a family of design concepts share common geometric configuration, structural performance, and manufacturing process characteristics.) The baseline concepts are those judged a priori to have the greatest potential for cost and weight savings, combined with acceptable development risk.

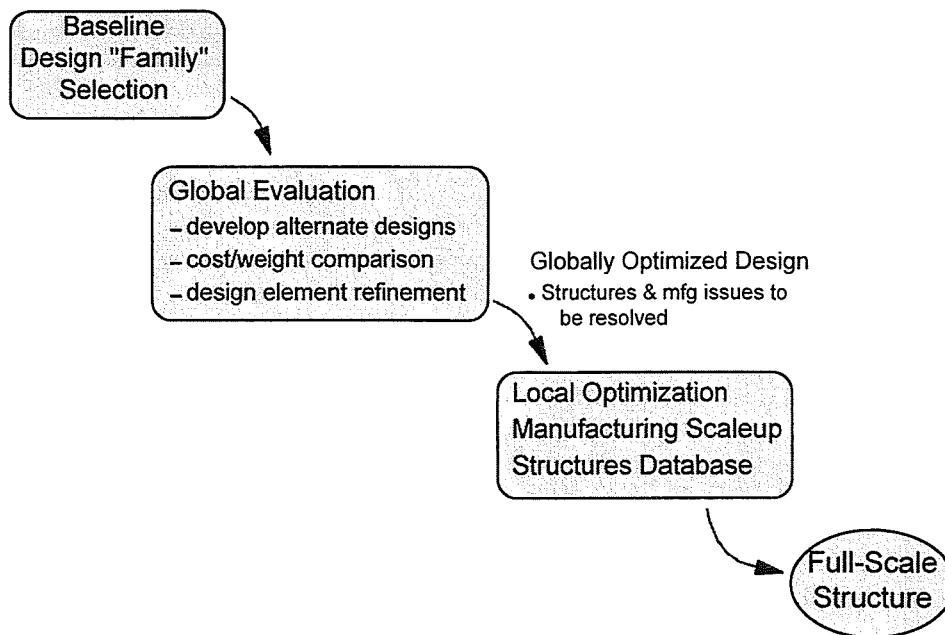


Figure 2-1. ATCAS Approach to Integrated Composite Fuselage Technology Database Development & Demonstration

During the second step, referred to as "global evaluation", preliminary designs are developed in sufficient detail to quantify significant cost and weight discriminators between the baseline

concepts and other potentially low-cost/low-weight concepts. Thorough cost and weight analyses are performed for each concept. New concepts are then generated within each design family by trading design variables in different combinations, leading to identification of an optimum design for each family and an understanding of the design details most significantly influencing its cost and weight. This step culminates with selection of a globally optimized design based on the cost and weight results, as well as an assessment of the risks associated with each concept.

The third step, termed "local optimization", takes the most attractive design identified in the second step and optimizes individual design elements (e.g., skin, core, frames, etc.) within the context of the design's total cost/weight efficiency. Significant attention is directed towards manufacturing scaleup and demonstration in addition to design refinement and structural evaluation.

This report documents global evaluation of the side panel region of a wide body commercial fuselage structure. (References 1 and 2 describe similar studies of the crown and keel regions conducted under the ATCAS program.) Structural concepts considered for the side panel included a skin/stringer design and a honeycomb core sandwich design. Manufacturing preplans were developed in conjunction with the design concepts as fabrication and assembly process assumptions have been shown to significantly influence cost results (Ref. 1). Major cost centers were evaluated to identify the "best" design details thereby improving the efficiency of each concept. Cost and weight results of these modified concepts were compared to a state-of-the-art metallic side panel structure. In addition, a risk assessment of each concept was performed which considered the value of potential weight savings. Finally, these results provided focus for the side panel local optimization activity which began immediately following completion of the side panel global evaluation.

3.0 TRADE STUDY GROUND RULES

3.1 Side Panel Definition

The subject of this global evaluation design study is the side panel of a wide body commercial transport fuselage. As depicted in Figure 3-1, the area of focus for this study was the fuselage barrel section located just aft of the main landing gear wheel well. This region of the fuselage, referred to as Section 46 on Boeing aircraft, has a constant diameter of 244 inches throughout the 398 inch length. The design parameters (i.e., geometric envelope, loads, and configuration constraints) are characteristic of an aircraft approximately 80 percent the size of a Boeing 747.

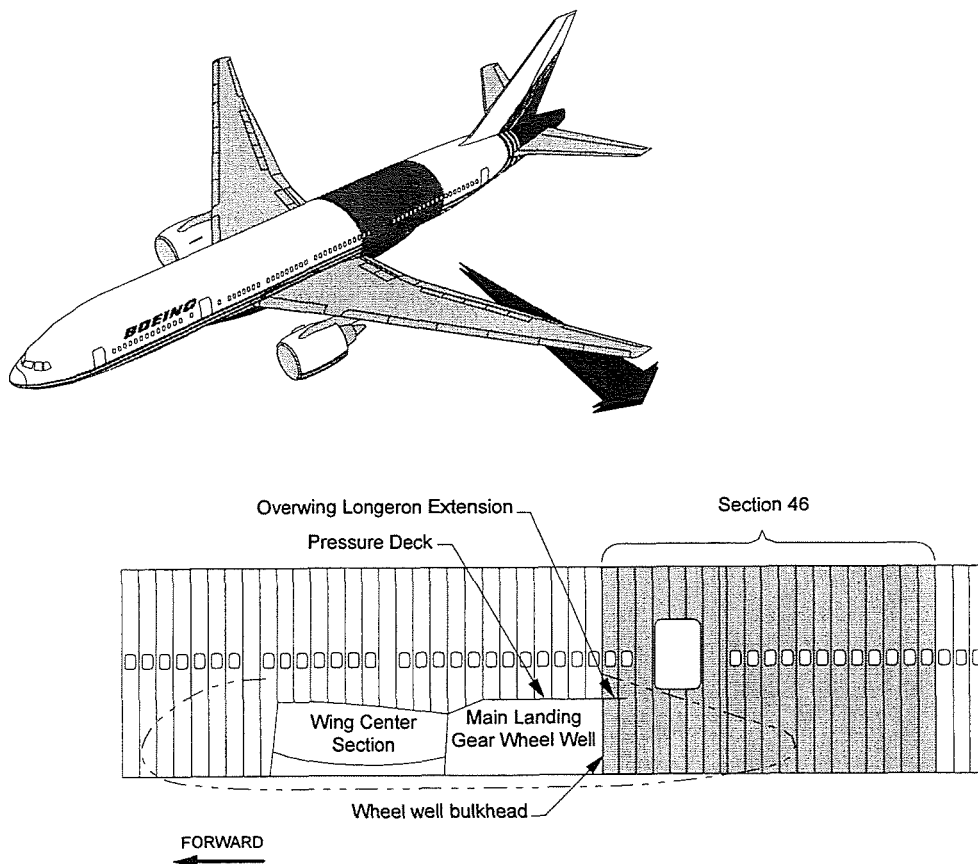


Figure 3-1. Location of Fuselage Section Considered in Trade Study

The ATCAS program baseline fuselage configuration is divided into four panels as illustrated in Figure 3-2. This definition resulted from balancing the manufacturing risk of building large cocured/cobonded assemblies with the cost efficiencies associated with larger panel sizes and reduced number of splices as compared to conventional metallic fuselage structure. Trade studies similar to that described herein have been performed on the crown and keel panels (Ref. 1, 2). Design features unique to the side panel are passenger doors, windows, a large cargo door,

and passenger floor structure. With the exception of the cargo door located on the right side panel, the two side panels share a common structural configuration. The ATCAS design development focused on the left side panel; however, issues associated with the cargo door were addressed through design of the passenger door structure. The design trades described herein encompass the side panel structural details (stiffened skin, circumferential frames, splices, cutout reinforcements, and passenger floor structure) only. Systems and interiors are assumed to be unchanged from those found on current transport aircraft.

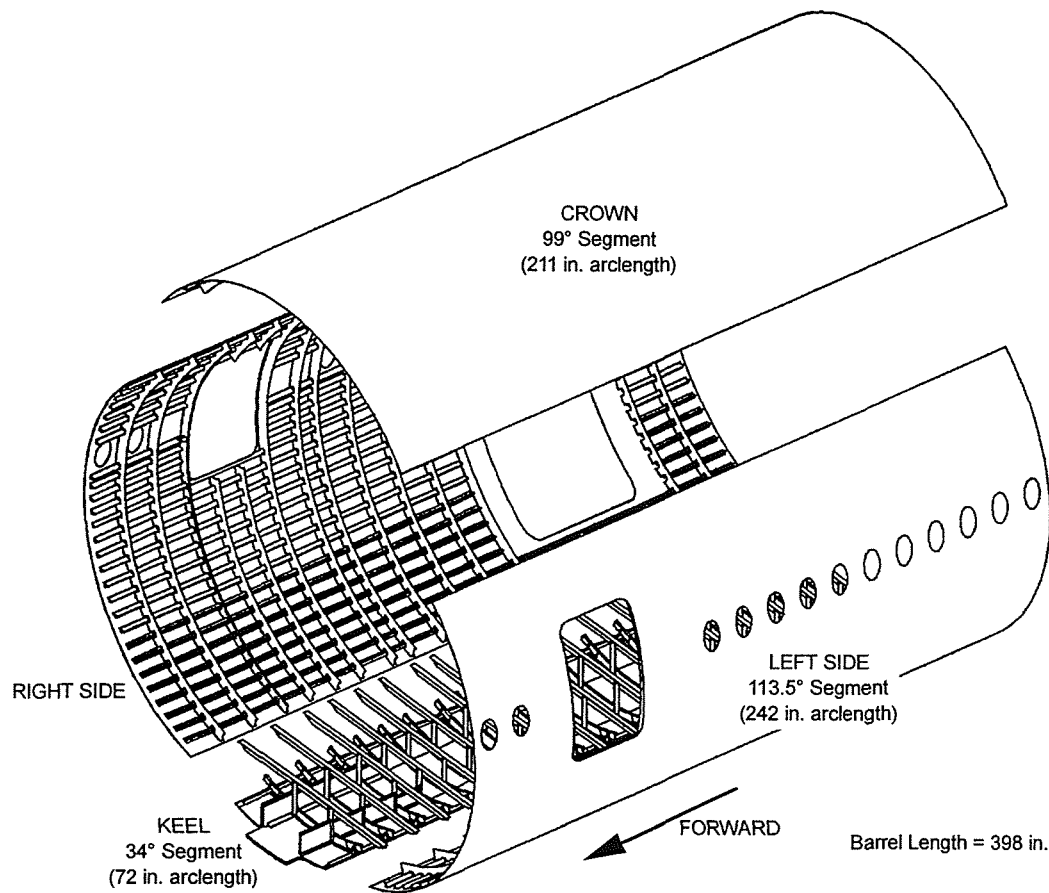


Figure 3-2. Exploded View of ATCAS Baseline Fuselage Panel Definition

3.2 Loads

Critical loads result from several flight and ground maneuvers. In general, maximum shear loads result from overall fuselage bending induced by pitch maneuvers while maximum axial tension and compression loads are induced by yaw maneuvers such as rudder, engine-out, or ground turn conditions. Furthermore, the aircraft configuration subjects the side panel to locally intensified load levels in several areas. As can be seen from the envelope contour plot of maximum axial load resulting from all flight conditions (Figure 3-3), the wheel well creates a load shadow in the lower forward area, with load intensity increasing as load is sheared in from the keel. Also, the

passenger door cutout and overwing longeron extension cause severe local load concentrations. Similar trends are observed for the maximum shear loads (Figure 3-4).

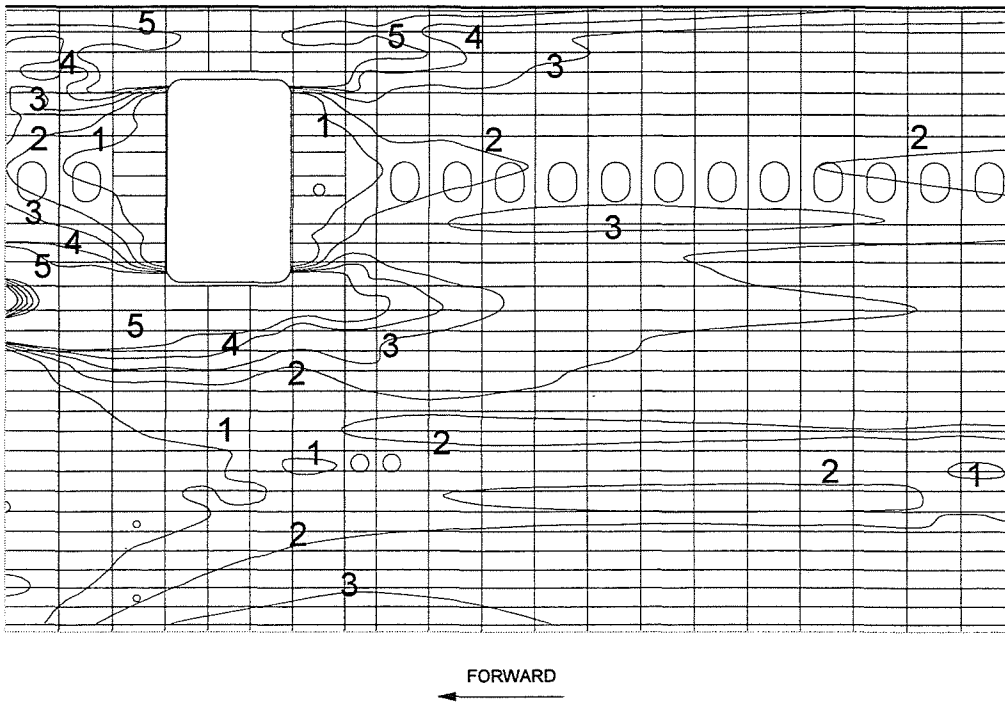


Figure 3-3. Side Panel Maximum Envelope Ultimate Axial Loads (kip/in)

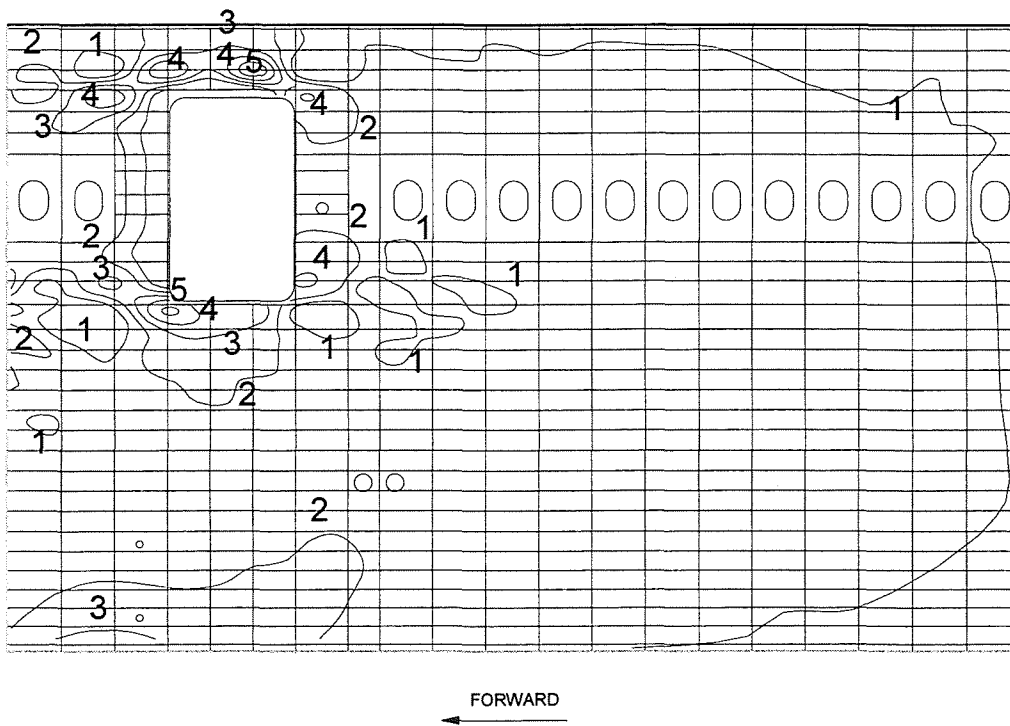


Figure 3-4. Side Panel Maximum Envelope Ultimate Shear Loads (kip/in)

Three ultimate load cases were used to size the semi-monocoque structure for strength and stability. The first includes flight loads combined with cabin pressure (9.1 psi, representing the maximum positive pressure differential), both factored by 1.5. The second consists of flight loads (multiplied by a factor of 1.5) acting alone. The third considers cabin pressure (18.2 psi, representing twice the maximum positive pressure differential) acting alone.

Several load cases were considered in sizing the circumferential frames, including ultimate pressure (18.2 psi), flight and ground loads, and crash loads. In addition, each of the flight load cases was evaluated in concert with cabin pressure. Similar conditions were used to design the passenger floor structure.

The side panel designs were also checked for two residual strength requirements under severe damage scenarios. The first requires limit axial load carrying capability in a panel with a transverse through-penetration severing one stringer and a length of skin up to twice the stringer spacing (or the equivalent thereof for sandwich configurations). The second condition consists of a 10.35 psi (1.15 times the normal operating pressure differential plus external aerodynamic pressure) pressure load acting on a panel with a through-penetration (oriented perpendicular to the hoop loading direction) severing a frame and up to two skin bays.

3.3 Manufacturing Considerations

Fabrication processes assumed for the side panel global evaluation study were characterized by a high degree of automation and integration of structural elements into cocured/cobonded assemblies. These trends were based on the results of the ATCAS crown and keel global evaluation studies which identified manual processes and high part count as significant cost drivers. Furthermore, it was assumed that a factory would be capitalized specifically for the selected manufacturing processes. This produced additional efficiencies due to optimum layout and utilization of automated equipment; these efficiencies were reflected in the cost estimating approach. The fuselage panel sizes (identified in Figure 3-2) were defined to maximize automated fabrication process efficiencies while minimizing the amount of assembly required. The individual panel configurations were also selected for optimum manufacturing efficiency and payoff. In the crown, for instance, a hat-stiffened skin design was selected. However, instead of attaching the stringer to the skin at the center of the middle segment of the hat as is common in hat-stiffened metallic structure, the cross-section was inverted and the two outer flanges used for skin attachment. This approach was much easier to tool and to achieve the required tolerances in the cured part. Similarly, the side panel sandwich design assumed that the honeycomb core could be machined to account for all facesheet thickness variations so that the inner skin surface had a constant radius. This greatly simplified fabrication and assembly of the circumferential fuselage frames to the skin panel, resulting in large cost savings over a non-tailored core design.

Four basic manufacturing processes were assumed for side panel fabrication. The skin panels, whether skin-stringer or sandwich, utilized automated fiber placement of individual plies directly onto the cure tool. Braided textile preforms consolidated using resin transfer molding (RTM) in matched metal tooling was identified as the most efficient process for large numbers of curved elements such as circumferential fuselage frames and window frames. Hot drape forming was

identified for thin to moderately thick straight parts such as stringers, floor beams, and stanchions. And finally, pultrusion was identified as a highly economical process for fabricating straight, constant cross-section elements required in large quantities such as stringers, floor beams, and intercostals.

3.4 Material Considerations

Material systems considered for the ATCAS side panel global evaluation were chosen based on availability, cost, performance, and compatibility with the selected manufacturing processes. In addition, those material systems considered for fabrication of ATCAS crown and keel panels were selected where possible to make use of previously developed technology and to minimize inventory in the theoretical production environment. The material systems considered in the trade study are summarized in Table 3-1 for the major side panel structural elements. Costs associated with each material were developed based on vendor data for projected production quantities for a wide body composite fuselage in the 1995 to 2005 time frame.

The baseline material for side panel fuselage skins consisted of a standard modulus fiber (Hercules' AS4) in a moderately toughened epoxy matrix (Hercules' 8552). Higher modulus fibers were not expected to be cost or weight efficient based on previous studies of the crown and keel panels (Ref. 1, 2). The toughened resin system was desired due to relatively high compression and shear loads. However, this desire was tempered by tension loads which dominate some regions of the side panel. For compatibility with the fiber placement process, a homogeneous (i.e., not interlayer particulate toughened) system was selected. As previous ATCAS efforts had observed superior large notch tension fracture performance in untoughened systems (Ref. 3), the lower cost Fiberite 938 untoughened resin system selected for crown panel skins was also evaluated in the side panel trade study. A nominal fiber areal weight of 190 g/m² provided economical layup of material in thick regions while retaining design flexibility in minimum gage areas.

Intermediate modulus carbon fiber (Hercules' IM6) in a toughened epoxy matrix was desired for stiffening elements, floor beams, and passenger floor stanchions due to high strength and stiffness requirements. Lower cost AS4 fiber was also evaluated for these elements and incorporated where appropriate. Straight elements such as stringers and sills were proposed to be drape formed from 8552 resin prepreg tape and fabric although pultrusion was also considered for elements with constant cross-section. Both pultrusion and drape forming processes were considered for the floor beams and intercostals. Curved elements such as circumferential frames were assumed to be braided and resin transfer molded. 3M's high performance PR-500 and Shell's low cost RSL 1895 resin systems were considered for resin transfer molding with PR-500 selected by Lockheed for fabrication of circumferential frames under their ACT contract (Ref. 4).

Lockheed selected woven AS4/AMD-0036 powder prepreg tow as the baseline material system for window frames (Ref. 5). Braided textile preforms consolidated using resin transfer molding (RTM) was retained as an optional process for these elements. Aluminum window frames typical of current production aircraft were also considered with the requisite isolation from the carbon/epoxy skin for galvanic corrosion protection.

Skins	AS4/938 (35% resin content) 12k tow AS4/938 (38% resin content) fabric AS4/8552 (35% resin content) 12k tow AS4/8552 (38% resin content) fabric
Stringers & Stanchions	IM6/8552 (35% resin content) 12k tape IM6/8552 (38% resin content) fabric
Core	8 pcf glass/phenolic HRP, 3/16" cell size
Core Closeouts	Injection molded epoxy filled w/ chopped graphite fibers AS4/RSL1895 triaxial braid T300/934 fabric
Fuselage Frames	AS4/RSL1895 triaxial braid AS4/PR-500 triaxial braid
Edge Frames	IM6/RSL1895 triaxial braid
Aux. Frames	AS4/RSL1895 triaxial braid IM6/RSL1895 triaxial braid
Sills	AS4/8552 (38% resin content) fabric
Intercostals	AS4/Epoxy (pultrusion compatible) IM6/8552 tape & fabric
Window Frames	AS4/AMD-0036 towpreg AS4/PR-500 triaxial braid 7075-T73 Aluminum
Floor Beams	IM6/Epoxy (pultrusion compatible) IM6/8552 (35% resin content) 12k tape IM6/8552 (38% resin content) fabric
Fasteners	Titanium Eddie bolts & Hilocks
Miscellaneous Fittings	6Al-4V Titanium

Table 3-1. Materials Considered in Side Panel Global Evaluation

Hexcel's HRP (0°/90° fiberglass-reinforced phenolic) honeycomb was chosen as the baseline core material for sandwich design concepts. The selection was based on keel panel cost/weight trade studies and impact damage resistance testing (Ref. 6). DuPont's Korex (aramid fiber reinforced phenolic) honeycomb which offers significant potential weight savings (at a cost penalty) was included in the design optimization trades discussed in Section 5.1.

Several materials and processes were considered for fabrication of sandwich core closeouts. Longitudinal panel splice closeouts were assumed to be braided and resin transfer molded tubular elements utilizing AS4 fiber and the RTM resin systems described above. Injection molding was chosen for circumferential splice and window closeouts. A chopped carbon fiber-filled epoxy was assumed to provide adequate stiffness to maintain fastener clampup for these applications. AS4/938 prepreg tape was selected for passenger door core closeouts due to their thin gage and large surface area.

Other materials included in the trade study included Narmco's Metalbond 1515-3M epoxy film adhesive (0.05 psf) for all bonded joints, selected due to Boeing's extensive experience with this

system for bonding 350°F cure composite structure. Due to complex attachment requirements in several locations, both titanium (6Al-4V) and aluminum (7075) were considered for various fittings. Use of the more expensive titanium avoided galvanic corrosion protection schemes required for aluminum.

3.5 Structural Criteria

The side panel design guidelines are related to structural requirements for ultimate strength, residual strength, stability, and warpage. While the side panel criteria were generally consistent with that used for the crown and keel global evaluation studies, advantage was taken of recent data generated under the ATCAS program on the materials of interest. Table 3-2 summarizes these guidelines.

Ultimate Strength	<p>basic: $0.0042 \text{ in/in} \leq e^{tu} \leq 0.0062 \text{ in/in}$ (material and laminate dependent) $0.0031 \text{ in/in} \leq e^{cu} \leq 0.0036 \text{ in/in}$ (material and laminate dependent)</p> <p>near stress concentrations: $0.0054 \text{ in/in} \leq e^{tu} \leq 0.0106 \text{ in/in}$ (material and laminate dependent) $0.0046 \text{ in/in} \leq e^{cu} \leq 0.0053 \text{ in/in}$ (material and laminate dependent)</p>
Damage Tolerance	<p>one structural unit failed to LIMIT load one cutout reinforcement element failed to LIMIT load axial skin stiffness 40% to 60% of configured panel axial stiffness laminates must contain at least 10% each of 0°, 45°, -45°, and 90° plies no more than four plies of same orientation grouped together</p>
Stability	<p>minimum wide column stability margin of 0.25 minimum stringer crippling margin of 0.25 skin buckling between stringers allowed at: 33% of ULTIMATE for skin thickness less than 0.11" 40% of ULTIMATE for skin thickness between 0.11" and 0.14" 100% of ULTIMATE for skin thickness greater than 0.14" minimum skin/stringer separation margin of 0.25</p>
Joints and Splices	<p>bearing/bypass requirements fastener countersink depth < 2/3 of skin thickness</p>
Panel Warpage	<p>cocured/cobonded element Poisson's ratio mismatch < 0.15 laminates must be balanced; any unsymmetry minimized and kept near laminate midplane</p>

Table 3-2. Side Panel Design Guidelines

The basic cutoff strain values for ultimate tensile and compressive strength, e^{tu} and e^{cu} , represent 0.25-inch diameter filled hole and open hole data at extreme environments, respectively, as a function of the material and laminate modulus, E_{lam} . The cutoff strain value for ultimate shear strength, e^{su} , was assumed to equal twice the tensile cutoff strain. A quadratic interaction criterion was used to account for combined loading. Isotropic stress concentration factors applied to ultimate panel loads were used to determine window and passenger door doubler

requirements. The strain limits for areas of stress concentration, such as at cutouts, are based on ultimate unnotched material capability under extreme environments. Properties for textile elements were assumed similar to those developed under the ATCAS crown frame development effort (Ref. 7).

Sizing for damage tolerance requirements was based on data generated for several material systems over a range of damage scenarios and structural configurations (Ref. 3, 8, 9) as part of the ATCAS crown study. This data was used to develop an empirical expression for residual strength of configured structure as a function of material, laminate stiffness, and damage size. This expression is

$$\epsilon_{all} = C_1 + C_2(2a) + C_3(E_{lam}) + C_4(2a)(E_{lam})$$

where ϵ_{all} is the allowable strain, C_1 through C_4 are empirically-determined material constants, $2a$ is the notch length, and E_{lam} is the laminate modulus (in Msi) perpendicular to the crack. This allowable strain is compared to the strain in the undamaged structure multiplied by a correction factor (Y) to account for load redistribution effects. For longitudinal notches (i.e., along the length of the cylinder), Y was assumed to equal one. For circumferential notches, Y was assumed to be 1.1 for skin-stringer designs as predicted in References 10 and 11 for sandwich designs.

Skin stability analyses utilized a curved orthotropic plate solution (Ref. 12) for compressive loading; for shear loading, an approximate flat plate orthotropic solution was employed with correction factors to account for curvature. An elliptic interaction equation accounted for effects of combined shear-compression loading (Ref. 13). No buckling of sandwich skins was allowed below design ultimate load, while post-buckled skins were allowed for the skin-stringer designs as indicated in Table 3-2.

For skin-stringer designs, an effective width approach was used to determine the strains in the skin and stringers at loads above the skin buckling load. The approach was developed for implementation in the COSTADE computer program (Ref. 14) and is similar to that outlined in Reference 13 for isotropic materials. The effective width, w , is related to the applied compression load on a single skin bay and stringer through the equation

$$Aw^2 + Bw + C = 0$$

where

$$A = \frac{PE_{sk}t_{sk}}{2p^2}$$

$$B = -E_{sk}t_{sk} \left[(D_{11} + D_{22})^{1/2} + D_{12} + 2D_{66} \right]_{sk}$$

$$C = -(AE)_{st} \left[(D_{11} + D_{22})^{1/2} + D_{12} + 2D_{66} \right]_{sk}$$

In these equations, E is the laminate axial modulus, t is the laminate thickness, A is the cross-sectional area, the D_{ij} are the laminate bending stiffnesses from classical laminated plate theory, and the subscripts sk and st represent the skin and stringer, respectively.

Euler column stability of the stringer and effective skin was assessed at design ultimate load including the effects of shear deformation of the column (Ref. 15). Stringer crippling was conservatively assumed to occur when the stringer buckled locally. Local stringer buckling was predicted based on flat orthotropic plate solutions (Ref. 16) with simply-supported boundary conditions assumed at each intersection between stringer subelements (e.g. between web and flange) and at each frame location. Skin-stringer separation was predicted using an empirical expression proposed by Northrop (Ref. 17, 19) which provided reasonable correlation with test data from the ATCAS crown database.

The joints and splices were sized according to bearing/bypass design values and minimum thickness requirements for countersunk fasteners. The load distribution in a joint with multiple rows of fasteners was determined based on elastic analyses. The resultant bearing loads and bypass strains were calculated for each fastener and compared to allowable curves given as a function of material and layup. These curves specify an allowable bypass strain for a given bearing load and were generated from available material properties and bolted joint data developed for the ATCAS crown design. For countersunk fasteners, the minimum skin thickness was taken to be 1.5 times the countersink depth to account for the effect of stress concentrations near the countersink and to ensure reasonable resistance to fastener pull-through.

The circumferential fuselage frames were sized with an assumed effective skin width of 21 inches (the nominal frame spacing) for loads not causing skin buckling (pressure acting alone and pressure combined with tensile flight loads). For flight loads producing skin buckling, a 5-inch effective skin width was assumed.

3.6 Cost Estimating Groundrules

Cost estimates generated in support of the side panel global evaluation utilized ground rules (Table 3-3) consistent with previous estimates completed for the ATCAS crown and keel panels. These ground rules are based on the recommendations of participants at NASA Advanced Composite Technology (ACT) cost workshops.

- | |
|---|
| <ul style="list-style-type: none"> • 300 shipsets with a peak rate of 5 shipsets/month • All new tooling is considered • Recurring labor wrap rate of \$100/hour • Nonrecurring labor wrap rate of \$75/hour • Part configuration per ATCAS engineering drawings • Part tolerances per engineering drawing or applicable material and process specifications • Cost estimates based on ATCAS manufacturing plans • Raw material costs based on ATCAS-supplied factors and vendor estimates • Estimated material cost is unburdened • Variance factor and improvement curves based on historical data • Does not include RDT&E or capital costs • Splice costs apportioned over the specific panels joined |
|---|

Table 3-3. Side Panel Cost Estimating Groundrules

The approach taken for the side panel estimates deviated to a certain degree from that followed in global evaluation of the keel and crown panels. A combination of detailed and parametric estimating methods were employed and for the first time on the ATCAS program, organizations external to Boeing contributed estimates for side panel elements. (Previous ATCAS trade studies relied on detailed Boeing estimates exclusively.) This approach reduced the time required to complete the estimates and did not compromise the validity of the results as the parametric estimates were correlated to previous detailed estimates of similar processes. Figure 3-5 describes the estimating methods and performing organizations for the various side panel elements.

		Non-Recurring Boeing Detailed	Recurring					
			Mat'l Boeing Detailed	Labor				
				Boeing Parametric	Boeing Detailed	Sikorsky Detailed	Lockheed Detailed	Northrop Detailed
SANDWICH	PANEL BOND ASSY.	●	●	●	●*	●		
	CIRC. FRAME FAB.	●	●	●		●	●	
	WINDOW BELT FAB.	●	●	●				
	DOOR REINFORCE. FAB	●	●	●				●
	PANEL ASSY.	●	●	●				
	Window Installation				●			●
	Door Installation				●			●
	INSTALLATION	●	●	●				
SKIN-STRINGER	PANEL BOND ASSY.	●	●	●		●		
	CIRC. FRAME FAB.	●	●	●			●	
	WINDOW BELT FAB	●	●		●		●	
	DOOR REINFORCE. FAB	●	●	●				●
	PANEL ASSY.	●	●	●				
	Circ. Frame Installation				●			●
	Door Installation							●
	INSTALLATION	●	●	●				

* Estimate considered fabrication and assembly of the core elements only.

Figure 3-5. Cost Estimating Methods Utilized For Side Panel Global Evaluation

4.0 DESIGN STUDIES

Global evaluation of the side panel was performed over an eighteen month period ending in February 1994. Design concepts were developed for two design families (see Appendix A). Family B consists of a skin/stringer/frame configuration with mechanically fastened circumferential frames, stringer clips, and J-section stringers cobonded to the laminate skin. Family D is a sandwich configuration with cobonded circumferential frames. Both designs included similar passenger door cutout reinforcement, window cutout reinforcement, and passenger floor support structures, as well as splices necessary to assemble the side panel into a complete fuselage structure. Note that the inclusion of a Family B design supersedes the side panel baseline Family C design selected early in the program (Ref. 1); this was a result of insight gained through the ATCAS crown and keel studies as well as a greater understanding of the side panel design drivers.

The following subsections describe each side panel design and corresponding manufacturing plans, design drivers, and cost/weight results. Additional information is provided in Appendices B and D which contain excerpts from the engineering drawings defining the Family B and Family D configurations.

4.1 Family B Design

The Family B side panel assembly is shown in Figure 4-1; representative engineering drawings are contained in Appendix B. The general side panel configuration, including location of passenger doors, window pitch, and structural hard points (e.g., the overwing longeron extension) was established at the airplane level and was not varied for this study. Due to the high shear loading encountered in the side panel, open section stringers were considered (in contrast to the hat-section stringers selected on the ATCAS crown panel) to facilitate a direct structural attachment at each stringer/circumferential frame intersection.

4.1.1 Family B - Design Description

Dominant design drivers for the Family B skin-stringer design are depicted in Figure 4-2. The design was sized to the criteria described in Section 3.5 with the axial damage tolerance calculations assuming a nominal skin penetration length of two stringer bays, which varied from 15.4 to 18.07 inches. Circumferential damage tolerance was checked for a penetration severing a frame plus the two adjacent frame bays of skin for a maximum damage length of 42 inches. Both hoop and axial damage tolerance were found to dominate the design of the side panel lower aft and middle regions (above and below the window belt). Moving down the panel, the axial compression loads peak at the lower longitudinal splice, requiring thicker skins and heavier stringers in the lower region to preclude panel buckling. Post-buckling was permitted (with skin-stringer separation as the limiting structural criteria - see Section 3.5), however, high combined shear plus axial compression loading resulted in thicker skin laminates than would have been required for high axial compression loading alone. The lower forward quadrant of the panel is in a load shadow caused by the main landing gear wheel well located in the fuselage section immediately forward of the study section. Shear buckling sized the structure in this area.

Typical stringer spacing is 7 to 9 inches, with structural stability and panel configuration (i.e., location of passenger door stops, door sills, panel splices, and the overwing longeron extension) being the chief constraints. A minimum gage constraint based on potential hail- and Zone 2 lightning-strike durability requirements was imposed but not found to drive the design. However, minimum gage constraints at countersunk fastener locations (e.g., the circumferential frames) did impact the design.

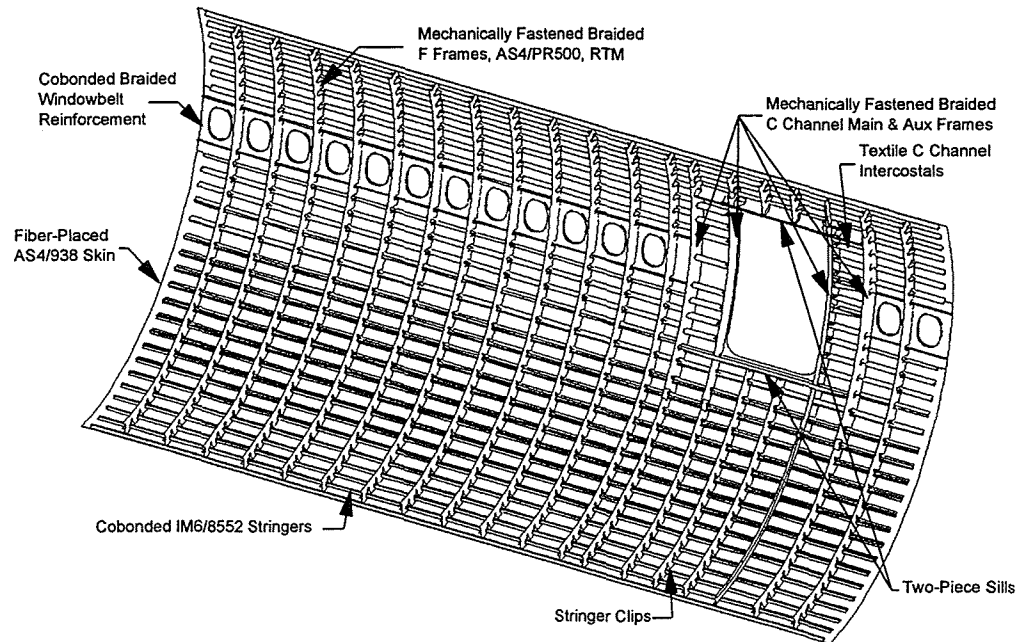


Figure 4-1. Family B (Skin-Stringer) Side Panel Assembly

The regions adjacent to the window cutouts are governed by ultimate strength due to the strain concentrations arising from maximum shear combined with axial and hoop loading conditions. The passenger door cutout skin doubler was also sized for ultimate strength concerns for maximum combined axial, shear, and hoop load conditions. Design of the circumferential fuselage frames was performed by the Lockheed Aeronautical Systems Company under a separate NASA ACT-funded program (contract number NAS1-18888). Frame design drivers included hoop tension due to cabin pressure and bending at passenger and cargo floor attachments. To simplify the design (as well as the fabrication trials), a constant cross-section design common to all locations was developed based on peak loads occurring at the passenger floor attachment a few stations aft of the passenger door. The frames all feature a fail-safe flange, necessitated by the mouseholes cut in the frame outer flange at each stringer location. The mechanically attached door edge and auxiliary frames, and upper and lower sills were designed by ultimate hoop and axial loads, respectively, as well as fail-safe conditions requiring limit load carrying capability with single structural unit failures. The mechanically fastened longitudinal and circumferential splices were sized by either bearing/bypass or minimum thickness required for flush head (countersunk) bolts. Stringers (at the upper and lower longitudinal splices) and frames (at the forward and aft circumferential splices) are included to

assure panel stability at each panel splice location. The area surrounding the overwing longeron extension is likewise sized by bearing/bypass concerns due to highly concentrated axial loads introduced into the side panel at this location.

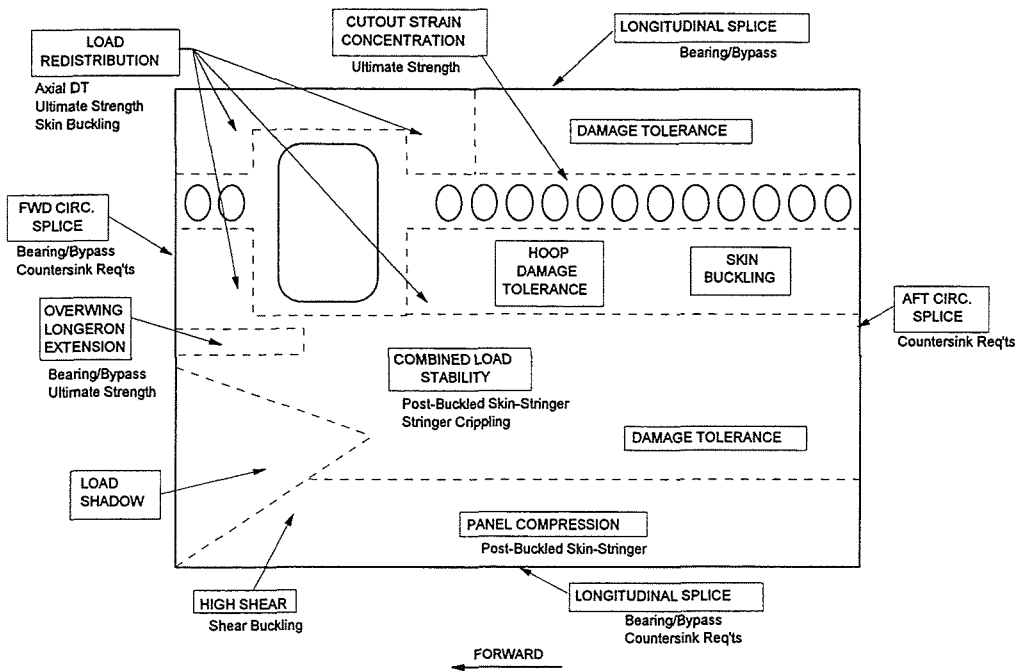


Figure 4-2. Skin/Stringer Critical Design Drivers

4.1.2 Family B - Manufacturing Approach

Manufacturing assumptions made in support of this study are consistent with those used throughout the ATCAS program. Namely, no constraints regarding factory size or capital requirements were considered. In addition, relatively near-term technologies (available within the next five to ten years) were assumed for the major fabrication processes. The following discussion describes the basic manufacturing processes identified for the Family B design, given in the order in which fabrication would occur.

The Family B design skin laminate is made from AS4/938 tow and fabric prepreg materials while the stringers utilize IM6/8552 tape and fabric prepreg materials. The fabric plies are limited to the inner and outer surfaces of the skin and stringer laminates to prevent fiber breakout during panel trimming and drilling of fastener holes. AS4/938 prepreg tow was used in the skin due to its large-notch tension damage tolerance characteristics as demonstrated in the ATCAS crown panel testing (Ref. 3, 8, 9), whereas IM6/8552 stringers provided slightly better value over AS4/938 (in terms of cost/weight efficiency) due to increased stiffness and higher small notch damage tolerance. AS4/8552 prepreg tow also was evaluated as a potential skin material system, but resulted in a heavier design due to its relatively poor large-notch tension damage tolerance as compared to AS4/938. The skin lay-up patterns, thickness, and stringer cross-sectional areas are shown in Figure 4-3.

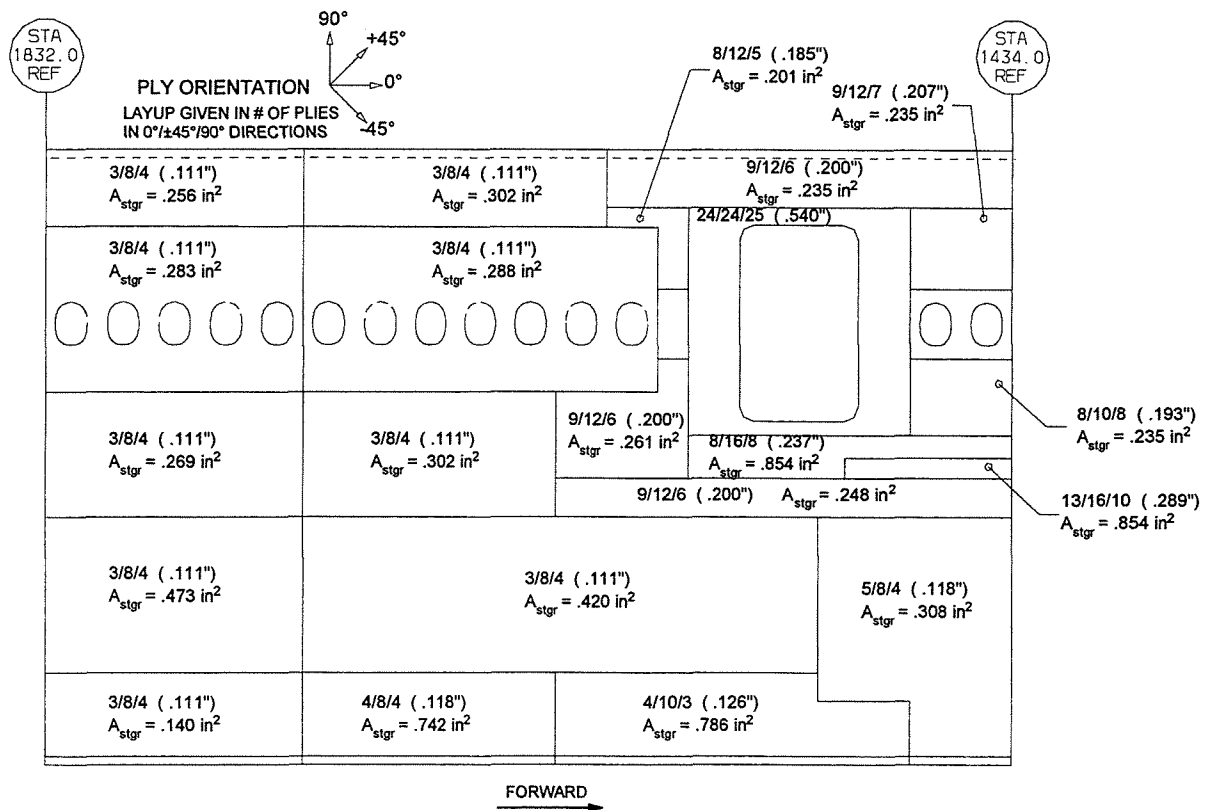


Figure 4-3. Design Family B Skin Thicknesses and Stringer Cross-Sectional Areas

The skin outer mold line (OML) and inner mold line (IML) fabric plies will be located with an automated fabric placement machine followed by a vacuum compaction cycle. Minimal hand working may be required to correctly position the fabric plies. The skin layup was assumed to use automated fiber placement with Tow Cut-and-Add (TCA) head technology; the skin will be fiber placed directly onto an OML INVAR cure tool. Projected fiber placement rates are 25 pounds per hour with a 5.74-inch wide band comprised of 32 individual prepreg tows. It was further assumed that the TCA head would fiber place over all cutouts except for the passenger door. The TCA head will be supported by an overhead gantry system similar to that shown in Figure 4-4. The OML tooling concept (Figure 4-5) was selected to avoid the complexity associated with a large number of high-tolerance tool components required on an IML tool to accommodate the cocured stringers and precured frames. One point in favor of an IML tool is the accurate, repeatable location of the cobonded elements to the skin panel.

All stringer cross sections are a "J" shape, except for the blade stringers located at the window belt and at the passenger floor water line. This later stringer provides attachment for the passenger floor with its outstanding flange angled relative to the side panel such that it is parallel to the floor. Blade stringers were considered early in the study for all stringer locations due to their lower manufacturing cost, but were eliminated due to poor crippling performance. The stringers do not drop plies within across the cross-section, but do drop plies in the lengthwise direction. The stringers are fiber placed into eleven distinct prepreg charges, formed with a hot drape process, cured on INVAR layup mandrels, and trimmed to produce thirty-three separate

stringer detail parts. Precured stringers were selected to reduce the panel cure tool complexity required for cocured stringers. To reduce the stringer tooling cost, they are cured as straight elements with no taper or joggles to account for skin panel thickness variation. This in turn restricted the skin laminate ply drops to a ratio of 300:1.

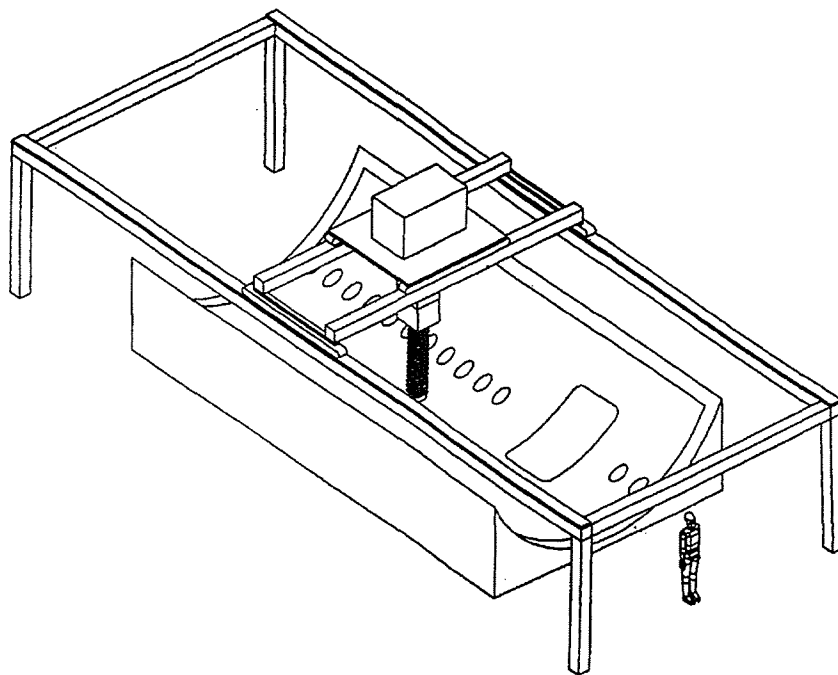


Figure 4-4. Automated Fiber Placement Machine

The integral window belt skin doubler/window frame element is a three-dimensional braided preform of dry AS4 fiber coated with powdered epoxy resin designed by Lockheed. As shown in Figure 4-6, this reinforcement structure incorporates a circular frame around the window cutout along with spanwise blade stiffeners immediately above and below the window. Radial tolerances will be critical to the fabrication process because all mating surfaces of the window belt detail must maintain intimate contact with the skin during cure for successful bonding. Due to the high bulk factor associated with the powder coated preform, several debulk cycles will be required. The window belt detail will then be cured using reinforced caul tooling prior to assembly on the skin panel.

The skin panel fabrication is completed with location of the pre-cured stringers and the integral window belt skin doubler/frame details. This is accomplished with the aid of a bond assembly jib which holds all the precured elements in position as shown in Figure 4-7 while film adhesive is applied to the mating surfaces. The entire jig is then rotated onto the uncured skin on the OML cure tool. No positive locating techniques (i.e., hard tooling pins) will be employed to hold the precured elements in position during cure due to concerns regarding cure shrinkage and panel warpage. The entire skin panel assembly is then autoclave cured. After cure, the panel is inspected using through transmission ultrasonic (TTU) methods to detect any anomalies such as porosity, imbedded foreign objects, incomplete bonds, or delaminations.

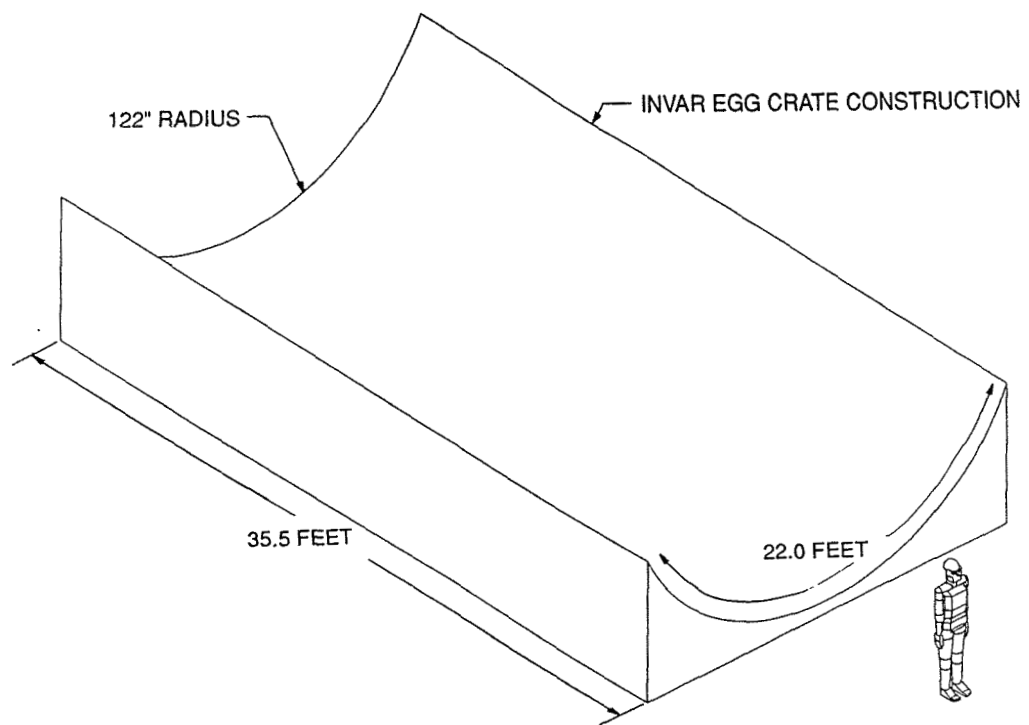


Figure 4-5. OML Cure Tool

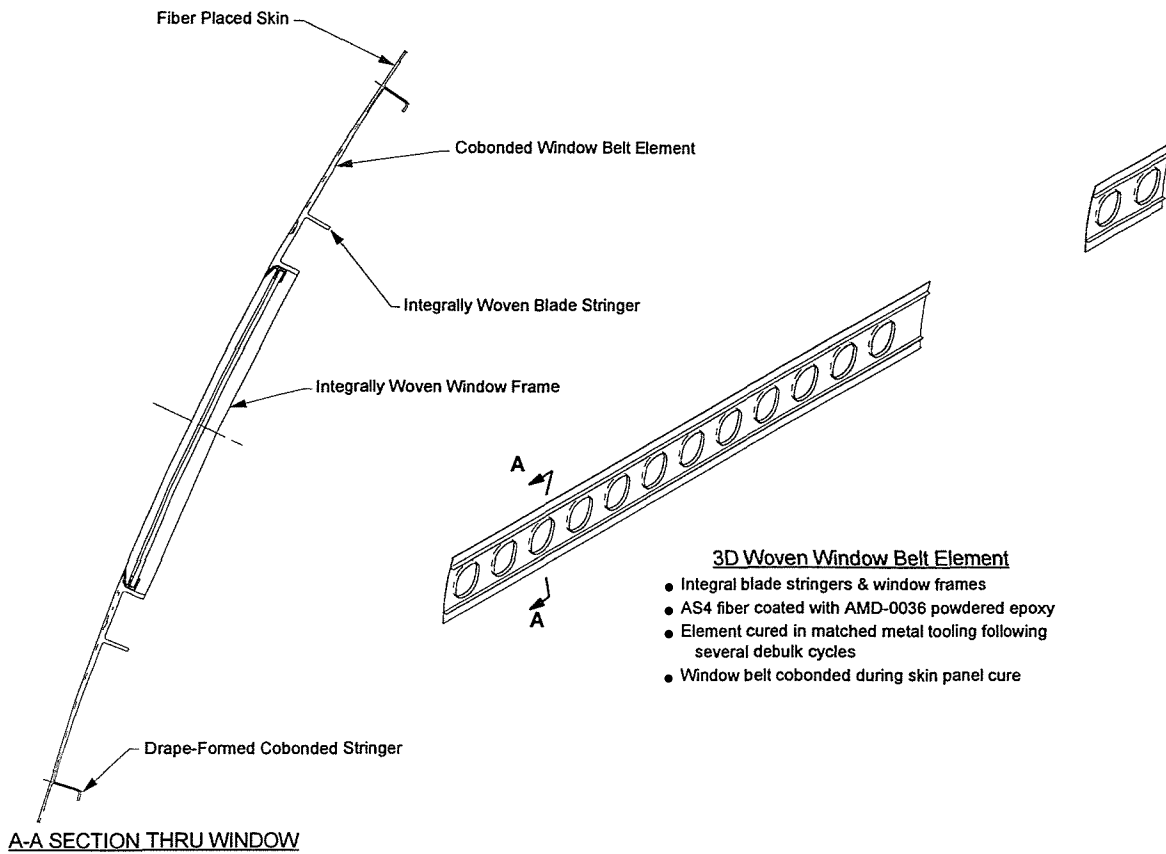


Figure 4-6. Window Belt Reinforcement Structure

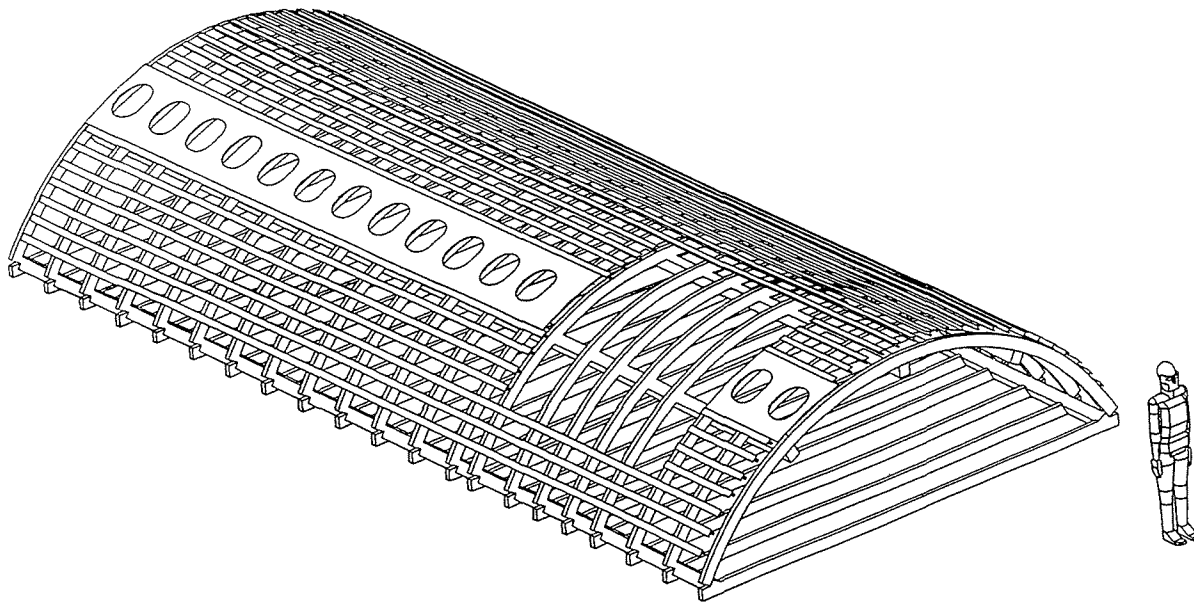


Figure 4-7. Bond Assembly Jig Used to Locate Precured Elements to Skin Panel

The circumferential frames are made from a two-dimensional triaxial braided preform of AS4 fiber which is resin transfer molded (RTMed) with PR-500 resin. The frame cross-section is an "F" shape with the skin attach flange joggled to match changes in the skin thickness. To create the preforms, numerous tows of graphite fiber will be braided around two solid mandrels in three successive passes. The two mandrels are mated vertically, then three more plies are braided over this stack. The braided preform, still on the mandrel, will then be placed inside a RTM tool set. Before the tool lid is placed on the tool set, the base plies are slit and folded outward to create the skin attach flange. Next, the lid is sealed onto the tool and the RTM process initiated. The part is fully cured within the RTM tool. The cured frame detail is removed from the RTM tool and trimmed away from the mandrel. Figure 4-8 shows these fabrication steps. Mouse holes are cut in the frame at each stringer location. Stringer clips, which provide a structural connection between the stringers and circumferential frames, are pultruded from a material with properties assumed equivalent to AS4/3501-6.

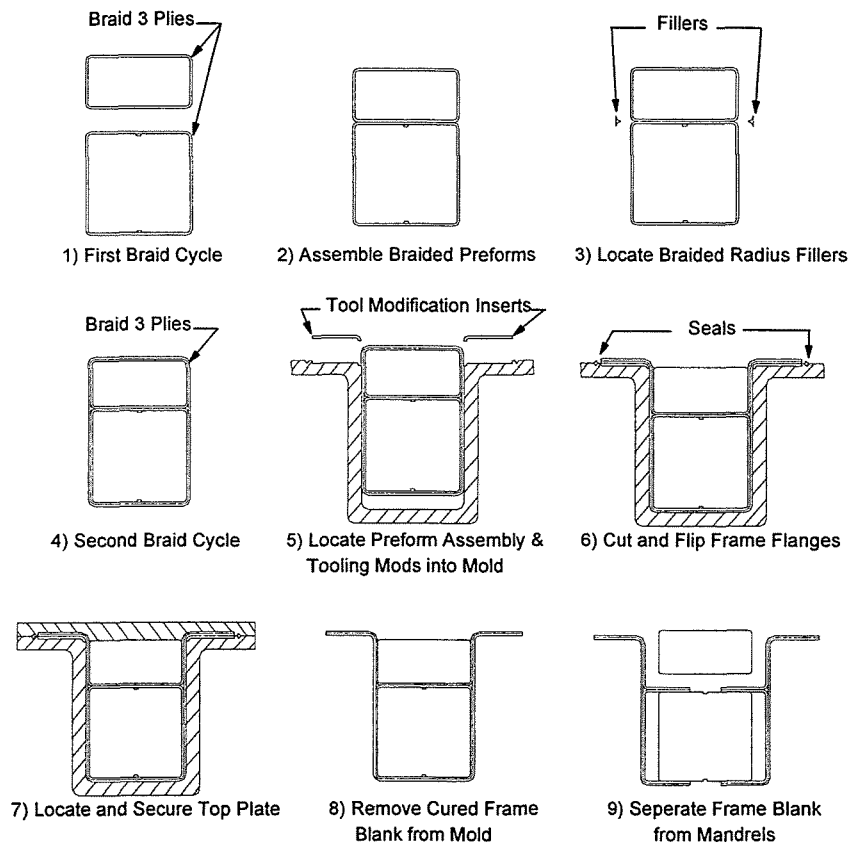


Figure 4-8. Circumferential Frame Fabrication Process

Assembly of the cured circumferential frames to the skin panel occurs in the following sequence. The frames are located into a Floor Assembly Jig (FAJ); liquid and solid shims are applied to the frame flanges per robotic measurements of the skin panel and frame attach flanges. The FAJ is positioned onto the skin panel where a semi-automated drilling and mechanical fastening process

attaches the frames to the cured skin. Stringer clips are positioned, drilled and mechanically fastened at each circumferential frame and stringer intersection. Mechanically fastening the frames to the skin was selected over cobonding (as selected on the ATCAS crown panel) to provide greater dimensional tolerance allowance on the skin-to-frame interface thereby loosening the tolerances which must be held on both the frames and skin panel. (The interaction of shear/compression loading combined with cabin pressure also was a deciding factor for mechanically attached frames, which provided a lower risk alternative to bonding.)

Figure 4-9 shows the door cutout support structure. Auxiliary and edge frames are made from the same material and process as the circumferential frames but with significantly heavier gages. The edge frames are a "C" cross section, and the auxiliary frames are a "Z" cross-section. The door sill inner chords are "L" sections made from a hybrid IM6/8552 tape and fabric laminate. The center sections of the outer chords feature integral flanges for stub frame attachment above and below the door. The door stop intercostals, also made from IM6/8552 tape and fabric, feature an integral skin attachment flange that joggles from the skin inner mold line over the edge frame flange. Door stop backup fittings are machined from a pultruded "T" section; the preform is a combination of tape and fabric using a material providing properties equivalent to AS4/3501-6.

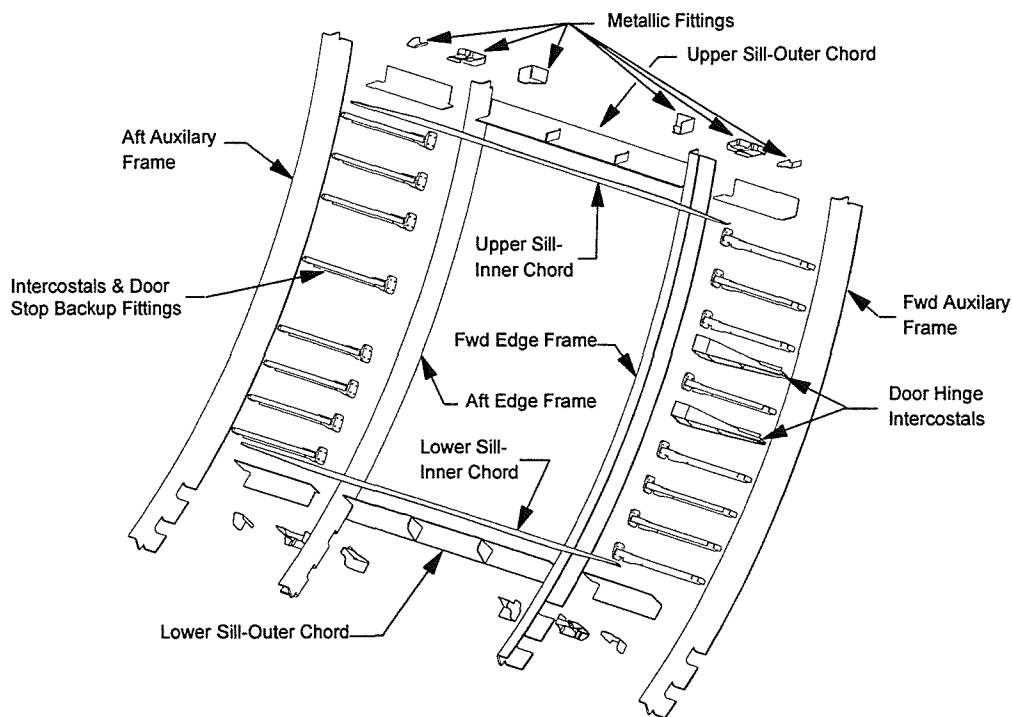


Figure 4-9. Door Surround Structure Assembly

After fabrication, the edge, auxiliary and sill details are mechanically fastened together to form an "eggcrate" assembly; this subassembly is then mechanically fastened to the skin panel. A combination of liquid and hard shims are applied at the skin to door structure interfaces where

necessary. Following this, the intercostals and stub frame splices are installed, again using mechanical fasteners.

Metallic fittings are used in locations where the nature of the loading (e.g., radius bending, short edge margins) renders composite laminate elements impractical. This includes locations such as door hinge intercostals, door sill to frame joints, stringer runouts into the main landing gear wheel well bulkhead, and cargo floor beam splices. Titanium (6Al-4V annealed plate) was originally specified in these applications to avoid the need for galvanic corrosion barrier fiberglass plies on the composite parts. However, after the initial cost estimates were generated, these elements were reconfigured to be aluminum with fiberglass isolation to realize a significant cost savings due to faster aluminum machining rates and lower raw material costs. Some representative metallic parts are shown in Figure 4-10.

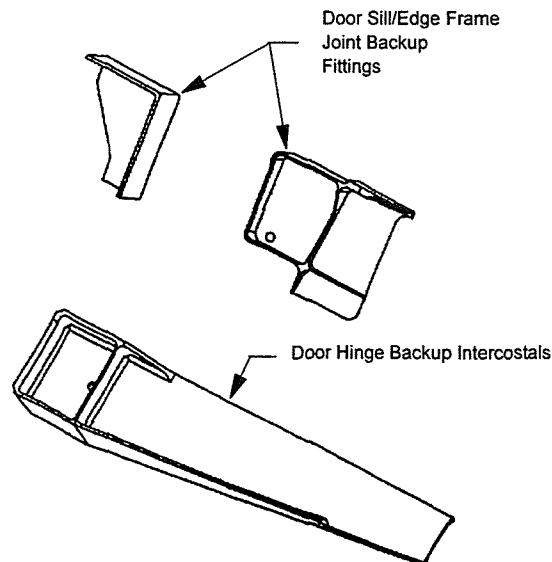


Figure 4-10. Typical Family B Metallic Fittings

Assembly of the Section 46 is accomplished in a FAJ which holds the panels in position while splice holes are drilled and fasteners installed. After the keel panel is loaded in the FAJ, the two side panels are located in position (Figure 4-11) and the skin lap splice fastener holes are drilled and the fasteners installed. A combination of solid and liquid shimming is used to accommodate any fit-up gaps at the splice. The circumferential frames are joined between panels with splice sections (cut from precured braided RTMed channels) using mechanical fasteners (see Figure 4-12). Passenger floor beams (pultruded from IM6/8552 tape and fabric in a "C" cross-section), passenger floor beam stanchions (which are a hot drape formed "Z" section with a joggle in the web), cargo floor beams, floor beam splices, and left- and right-hand overwing longeron extensions are installed next as depicted in Figures 4-13 through 4-15. Assembly of the barrel section is completed with installation of the crown panel using lap splices as was done for the side-to-keel panel splices.

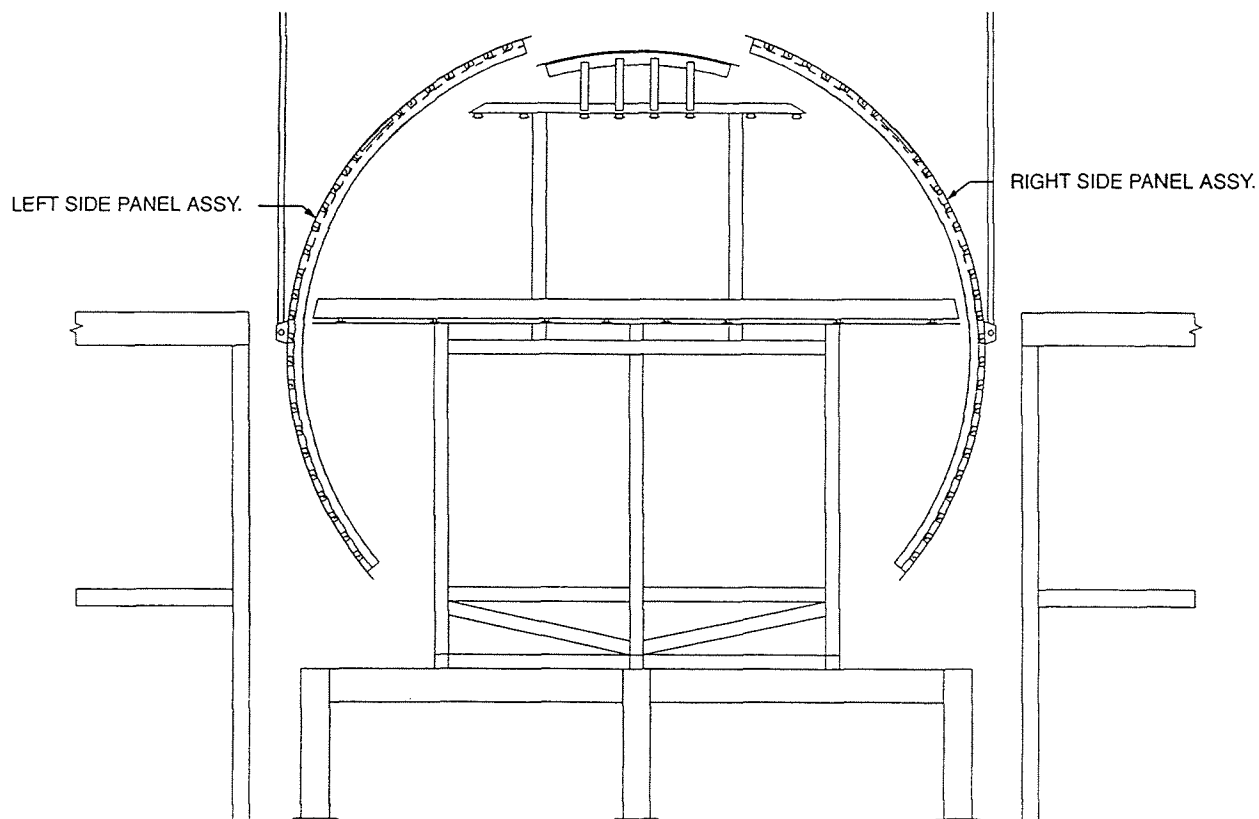


Figure 4-11. Section 46 Assembly Process Showing Positioning of Side Panels

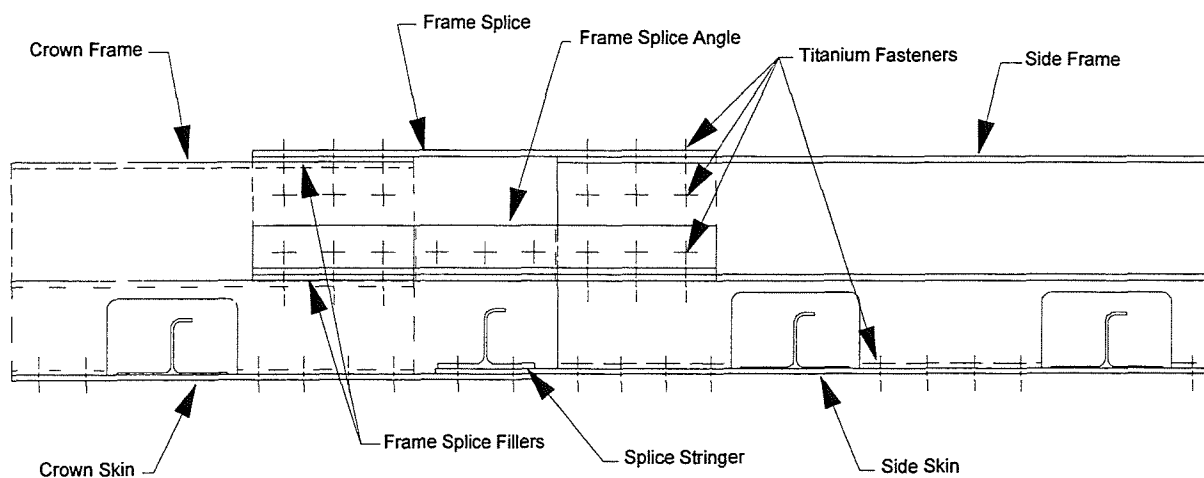


Figure 4-12. Longitudinal Panel Splice

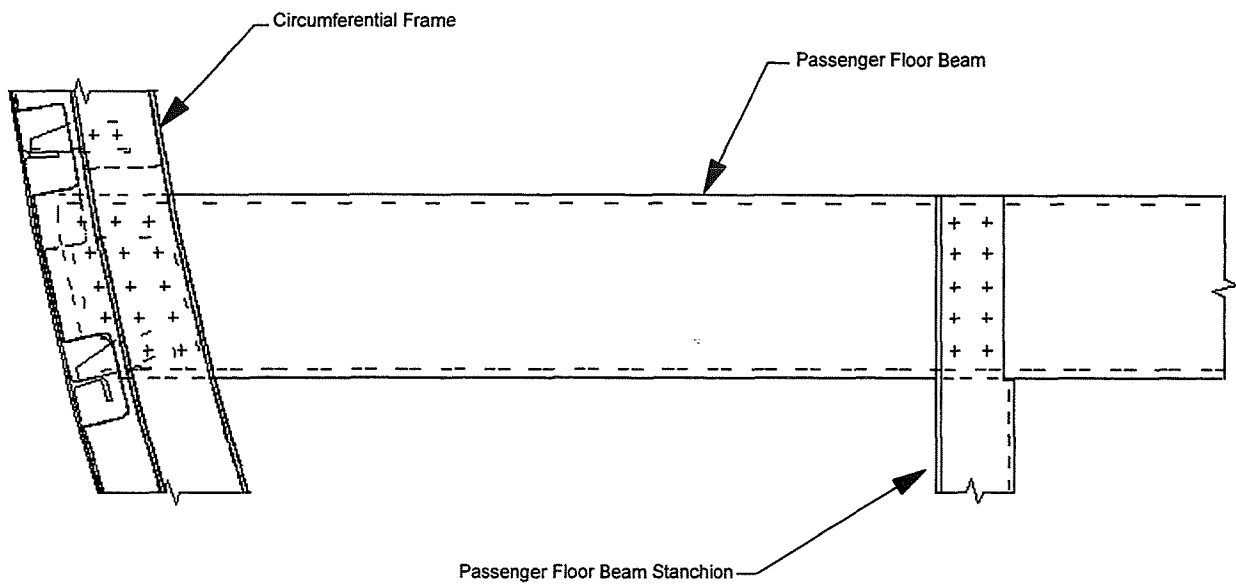


Figure 4-13. Passenger Floor Beam and Stanchion Installation

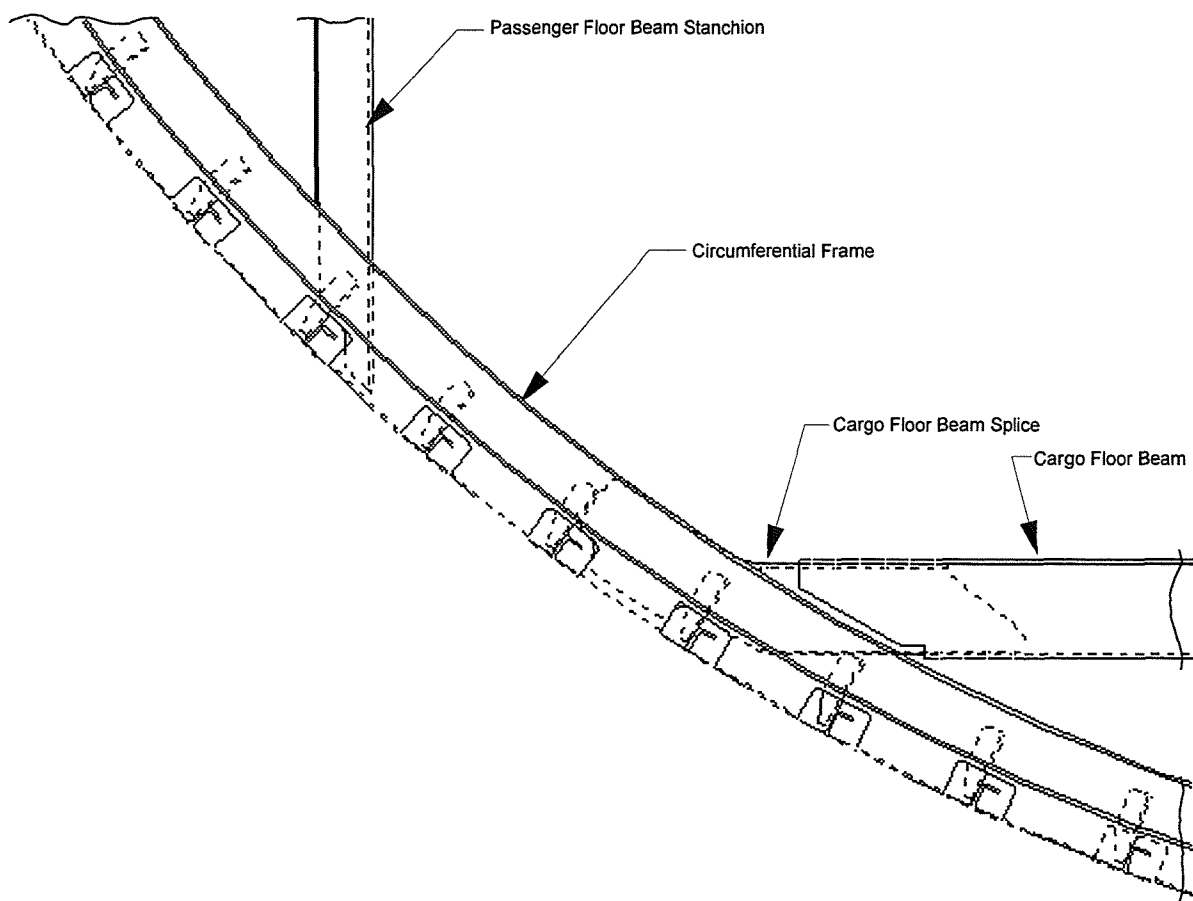


Figure 4-14. Cargo Floor Beam and Passenger Floor Stanchion Installation

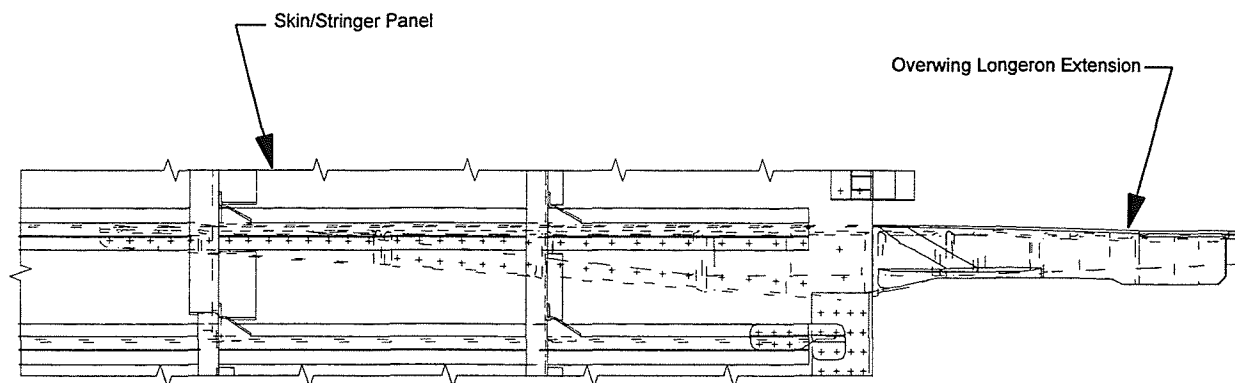


Figure 4-15. Overwing Longeron Extension Installation

Joining of Section 46 to the sections forward and aft is conducted through a butt-splice configuration. The completed fuselage sections are brought into position in a FAJ. Internal and external circumferential splice straps are located in place; any gaps are shimmed up to allowable limits using a combination of solid and liquid (moldable) shim materials. The splices are then drilled and mechanical fasteners installed; stringer splice fittings are similarly located, shimmed and fastened. Titanium eddy-bolts are used throughout the Family B design for mechanical assemblies.

4.1.3 Family B - Critical Issues Identified

The design study described above revealed several structural issues which must be resolved prior to proceeding with the Family B design in a production environment. These include verification of panel stability under combined compression-shear loading with cobonded open section stringers, stringer termination at the passenger door, and validation of the fuselage frame mouse-hole detail. Experimental verification of skin-to-stringer bond integrity under combined shear and compression loading must be demonstrated as well as performance and inspectability under fatigue conditions. Analytical and experimental verification of design details associated with large cutouts is also required to maximize the efficiency of this large cost center for the side panel.

Manufacturing issues for the Family B skin-stringer side panel are magnified by the large panel size. Demonstrating repeatable accurate location of the cobonded window belt and stringers will be critical to successful fabrication of the skin panel assembly. Resin transfer molding process uniformity and repeatability are concerns for the 20-foot long fuselage frames, as well as validating the flexible tooling concept which was assumed to reduce tooling cost through the use of removable inserts. Cost-effective layup of the 600 square foot skin laminate charge must be demonstrated, particularly in light of the design complexity associated with local fiberglass protection plies and provisions for systems attachments not explicitly addressed in this study. Also, assembly methods must be validated to ensure repeatability and reliability to maximize the efficiency of this large cost center. Finally, both production and in-service quality assurance procedures must be demonstrated. Overall, the Family B design presents lower manufacturing risk than the Family D design.

4.2 Family D Design

A single sandwich panel configuration, designated Design D1, was developed as depicted in Figure 4-16; representative engineering drawings are located in Appendix C. As for the Family B design, the general side panel configuration, including location of passenger doors, window pitch, and structural hard points (e.g., the overwing longeron) was established at the airplane level and was not varied for this study. The circumferential frame spacing was also fixed for the initial study, however the design optimization phase of the side panel global evaluation considered weight and cost advantages of increasing the frame spacing. Finally, to reduce manufacturing complexity (and hence cost) associated with circumferential frame fabrication and assembly, the sandwich shell was constrained to a uniform thickness throughout the entire panel.

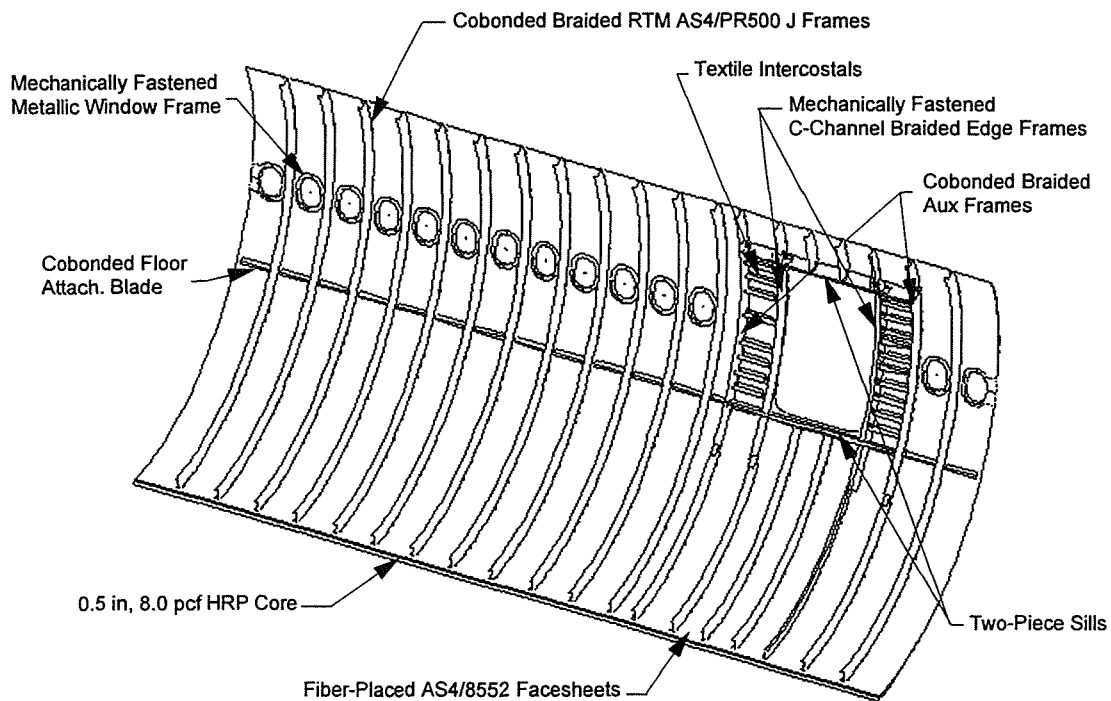


Figure 4-16. Family D (Sandwich) Side Panel Assembly

4.2.1 Family D - Design Description

The Family D critical design drivers are illustrated in Figure 4-17. The criteria described in Section 3.5 was used for structural sizing with the axial damage tolerance calculations based on a nominal skin penetration length of 14 inches. (This dimension, representing large rogue damage, was selected for consistency with the Family B design criteria.) As for the Family B design, circumferential damage tolerance was checked with a penetration severing a frame plus the two adjacent frame bays of skin for a maximum damage length of 42 inches. Hoop and axial damage tolerance dictated the design of the majority of the aft two-thirds of the panel. The compression loads increase in the lower side, requiring thicker facesheets to prevent panel buckling. (The

manufacturing constraint to maintain a constant sandwich thickness over the entire panel forced a compromise between local structural efficiency and overall panel weight savings where the off-optimum core height plus thicker facesheets in the lower side area carried a smaller weight penalty than increasing the core thickness over the mid and upper regions of the panel.) The forward lower region is in a load shadow caused by the main landing gear wheel well bulkhead; shear buckling sized the structure in this area. An eight-ply outer facesheet minimum gage criteria was also imposed to accommodate potential hail- and Zone 2 lightning-strike durability requirements. Due to the added trade study constraint to maintain equivalent inner and outer facesheets, this criteria drove the design in lower loaded areas such as the panel lower forward and upper aft corners. Finally, one structural ply of plain weave graphite fabric was specified on the outer surface of each facesheet to preclude fiber breakout during drilling or trimming operations and on the inner facesheet surfaces against the honeycomb core to prevent migration of the core details during skin panel cure.

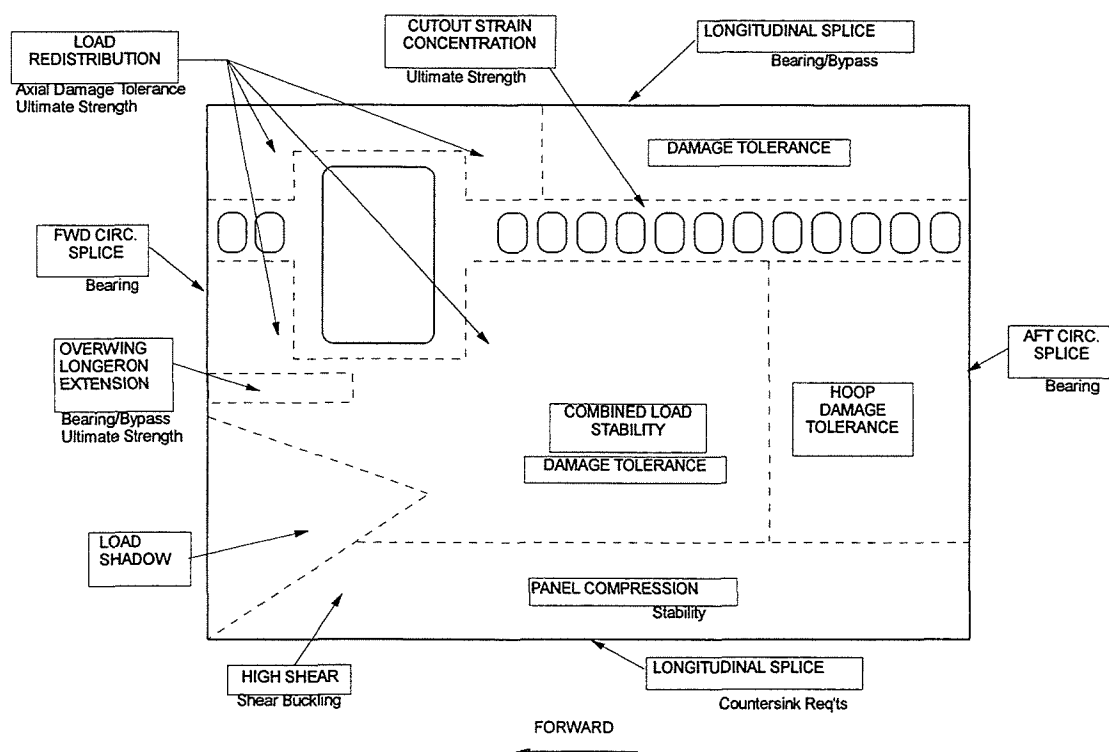


Figure 4-17. Sandwich Critical Design Drivers

The regions adjacent to the window cutouts are governed by ultimate strength due to strain concentrations under maximum shear combined with axial and hoop loading. The passenger door cutout skin doubler was also sized for ultimate strength under combined axial, shear, and hoop load conditions. Higher small-notch ultimate strength of the AS4/8552 facesheet material allowed 20 to 30 percent thinner laminates in these areas than was possible for the Family B design. Circumferential frame design drivers were identical to those for Family B, however, a fail-safe flange was not required because the sandwich design did need mouseholes. Because of

the higher bending stiffness afforded by the sandwich construction and the absence of mouseholes, the required fuselage frame cross sectional area was approximately 10 percent less than the Family B frames. The passenger door edge and auxiliary frames, upper and lower sills, and door stop intercostals were sized to the same ultimate and fail-safe conditions as the Family B door structure and hence were quite similar. The longitudinal and circumferential splices were sized by either bearing/bypass strength or minimum thickness required for countersunk mechanical fasteners. A mechanically fastened frame was located at the forward and aft circumferential splices to ensure a boundary is formed to prevent panel buckling through the splice. Bearing/bypass concerns also sized the area surrounding the overwing longeron extension due to highly concentrated axial loads at this location.

4.2.2 Family D - Manufacturing Approach

Manufacturing assumptions made in support of this study are consistent with those used throughout the ATCAS program. Namely, no constraints regarding factory size or capital requirements were considered. In addition, relatively near-term technologies (available within the next five to ten years) were assumed for the major fabrication processes. The following discussion describes the basic manufacturing processes identified for the Family D design, given in the order in which fabrication would occur.

The design utilizes nominally 0.5-inch thick 8.0 pound per cubic foot fiberglass/phenolic (HRP) honeycomb core (3/16-inch cell size) while the facesheets are AS4/8552 with a nominal resin content of 35 percent by weight. Both of these material systems were selected for compatibility with the ATCAS keel panel for which extensive sandwich panel material screening evaluations were performed (Ref. 19). The design features constant sandwich panel thickness achieved by varying the nominal core height to accommodate variations in facesheet laminate thickness. The core thickness ranges from 0.500 inches in the lower mid section to 0.581 inches in the upper aft section, thus maintaining a constant 0.701-inch panel thickness.

The facesheets will be fiber placed (using TCA technology) directly onto a female OML INVAR cure tool (Figure 4-5). As for the Family B design, projected fiber placement rates are 25 pounds per hour with a 32 tow, 5.74 inch bandwidth. Based on a trade comparing fiber placement efficiency against material scrap costs, all cutouts except for the passenger door will be trimmed after panel cure. The TCA head will be supported by an overhead gantry system similar to that shown in Figure 4-4. The facesheet inner and outer surface fabric plies will be positioned with an automated placement machine followed by a vacuum compaction cycle; minimal hand working was assumed to properly locate these plies. The inner and outer facesheet layup pattern is shown in Figure 4-18. Selection of the OML tooling concept followed the same rationale as for the Family B design.

Following trimming to near-net thickness, the individual honeycomb core details will be heat formed to match the outer facesheet inner surface. The formed core pieces and the core closeout details will be assembled into a core blanket (Figure 4-19) on a core assembly jig. After a single ply of AS4/8552 plain weave 0°/90° fabric and a ply of film adhesive are placed on the jig surface, each core detail will be positioned onto the jig. Foaming adhesive will be located at all core

splices and the entire assembly oven-cured. After final core machining, the cured core blanket will be transferred to the OML skin cure tool for assembly with the facesheets.

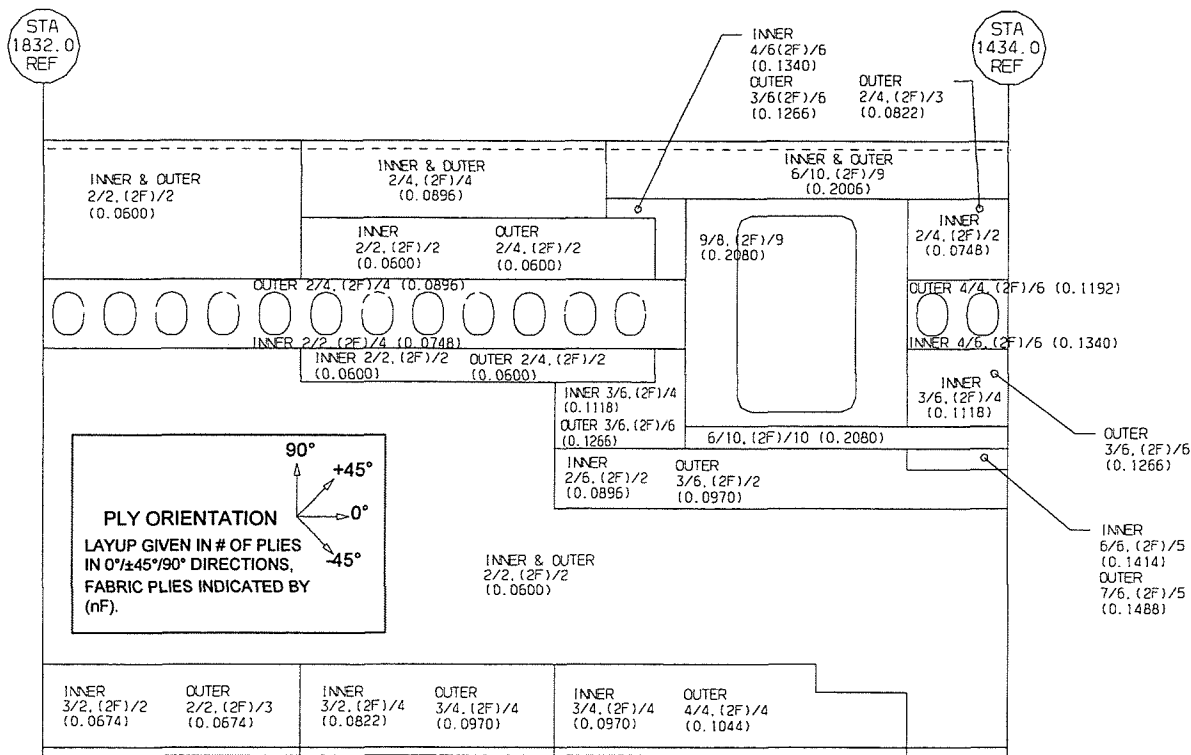


Figure 4-18. Skin Panel Layup Configuration

The circumferential frames have a "J" cross-section as the fail-safe flange is not required for the Family D design. The skin panel constant inner mold surface radius allows for a common frame detail to be located at all frame stations except locally around the passenger door cutout. This significantly reduces part count and frame fabrication, tooling, and installation cost.

Like the Family B design, the frames are fabricated by resin transfer molding a triaxially braided dry preform. To create the frame preform, 12k tows of graphite fiber are braided around a solid mandrel in a total of five passes through the triaxial braider. The preform, still on the braiding mandrel, is placed inside a RTM tool set. Before the tool lid is placed on the tool set, the base plies are slit and folded outward to create the flange for the frame base. Next, the lid is sealed onto the main tool and the RTM process is initiated. The part is fully cured within the RTM tool. The cured preform is removed from the tool and trimmed away from the mandrel. Figure 4-20 illustrates this process. Lastly, the part is final trimmed and inspected.

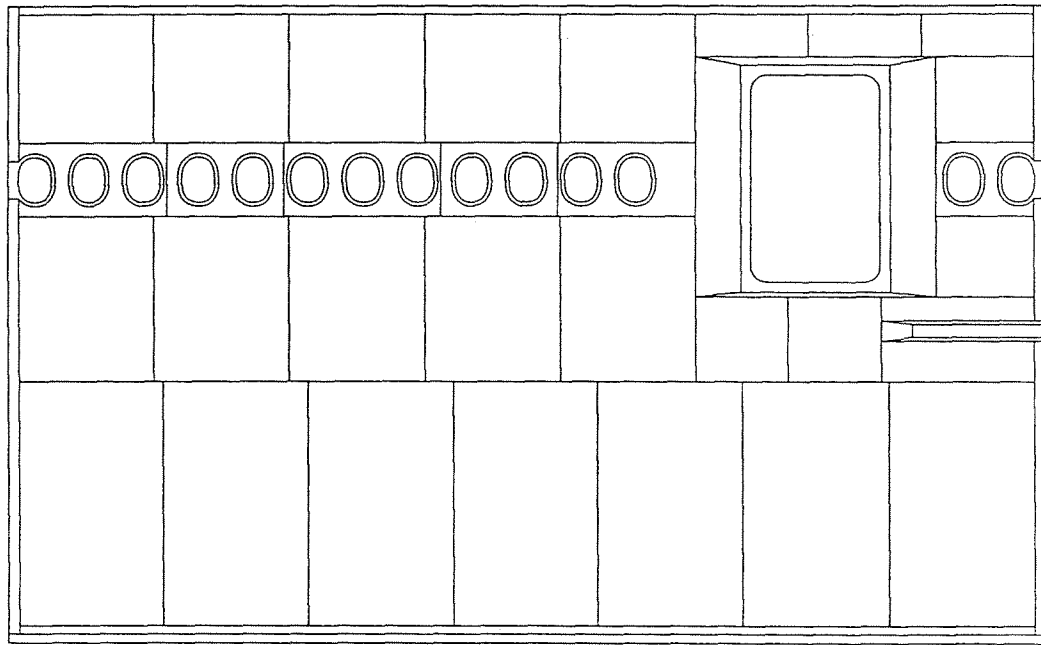


Figure 4-19. Assembled Core Blanket

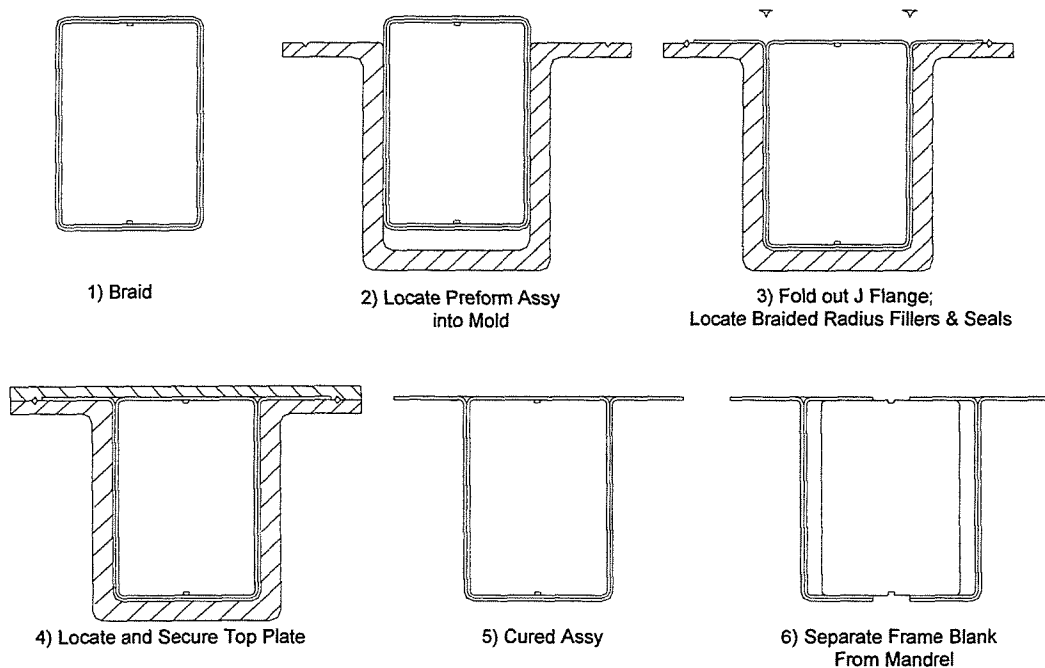


Figure 4-20. "J" Frame RTM Fabrication Process

Full depth edge bands were selected for the side panel perimeters as well as around the window cutouts. The edge bands avoid (a) the complexity associated with fabrication of core ramps and (b) the cost associated with automated fiber placement of ramped facesheet contours while providing a closeout for the core designed to resist penetration of contaminants from the external

environment. Closeout materials were selected to provide the strength required for mechanically fastened attachments such as window frames, panel splices, aircraft systems attachments, and pressure shell penetrations. Window and circumferential splice closeouts are fiber reinforced injection molded solid epoxy details which aid in the splicing and attaching of details with mechanically fasteners. The longitudinal splice members are made of three tubular textile preforms that are precured into a single rectangular box configuration that runs the entire length of the panel edge as indicated in Figure 4-21. The tubular concept was selected to provide maximum weight efficiency with sufficient stiffness to react fastener clampup for the four-row longitudinal panel splice.

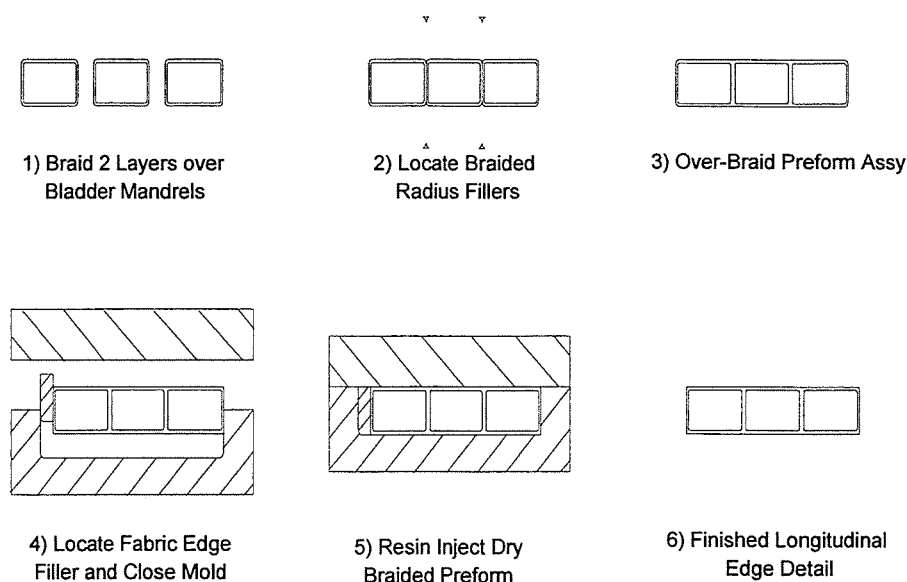


Figure 4-21. Longitudinal Panel Splice Closeout Fabrication

The skin panel fabrication is completed with positioning of the assembled core blanket with closeout details and film adhesive between the facesheets, followed by location of the circumferential frames. The frames are loaded in a FAJ, film adhesive applied to the attach flanges, and the FAJ rotated into position on the sandwich panel. The ends of each frame are held in position by a set of tools attached to the OML cure tool. The resulting assembly is shown in Figure 4-22. Reinforced rubber cauls under a silicon vacuum bag are applied and the entire panel autoclave cured. Cobonding of the circumferential frames avoids fasteners penetrating the sandwich structure with the accompanying potential for fluid penetration.

The window frame consists of a mechanically fastened aluminum forging (Figure 4-23) (similar to those currently used on current production transport aircraft), selected to provide a cost/weight comparison with the Family B textile integral window belt doubler and frame. (Note that this metallic window frame was not optimized for the composite sandwich panel design.) The fasteners penetrate the honeycomb panel at the solid core closeout in order to maintain integrity against fluid ingress. Fiberglass plies cocured on the panel OML and IML surfaces adjacent

to each window cutout provide galvanic isolation between the aluminum frames and the graphite skin; these plies are incorporated into the skin layup sequence.

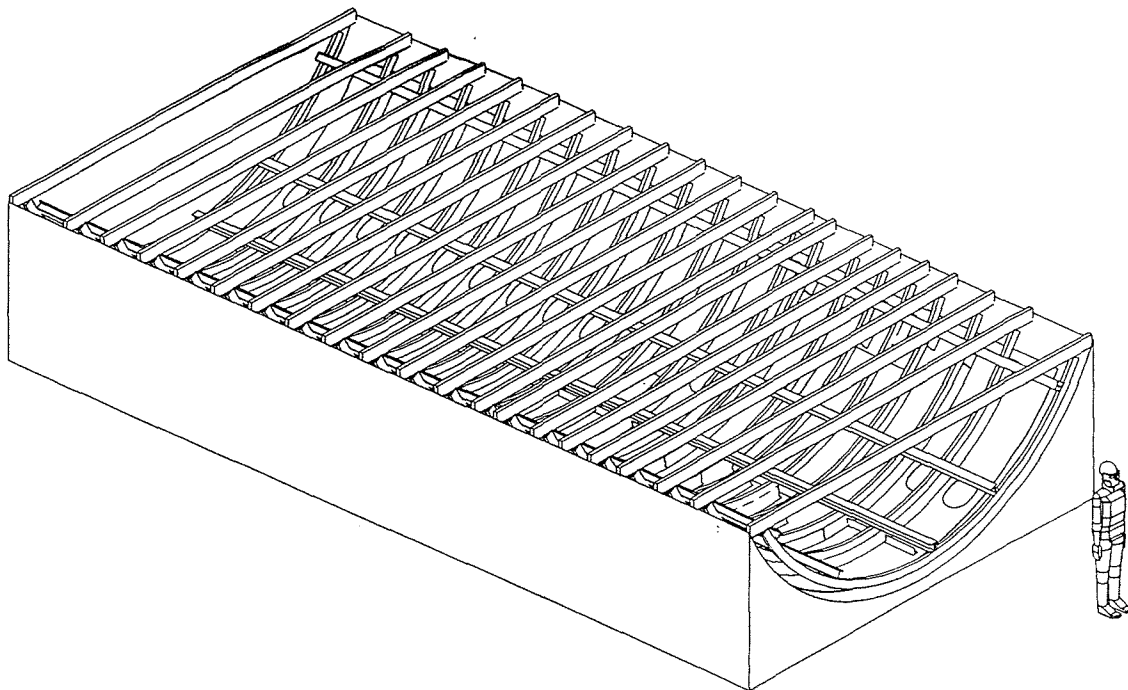


Figure 4-22. Assembly of Precured Details to Sandwich Panel

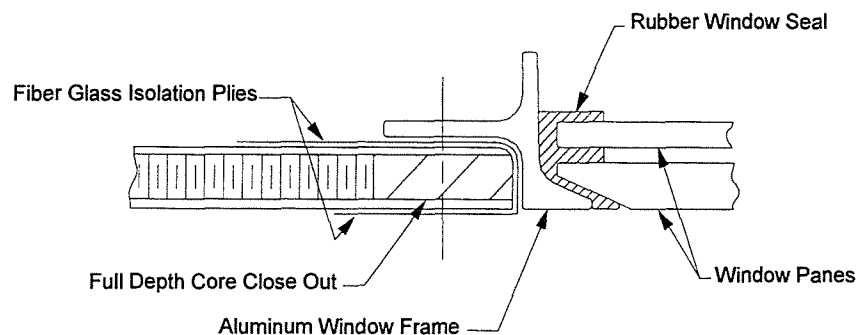


Figure 4-23. Window Reinforcement Structure

At the passenger door, because of the high potential for in-service damage, the edge frames are mechanically fastened, as are the intercostals and sills. Tight tolerances associated with door rigging and operation also drives the use of mechanical fastening. The skin panel transitions from sandwich-stiffened to a solid laminate due to the thickness required at the cutout and to allow for the mechanically fastened door surround structure. The door cutout reinforcement structure is quite similar to that of the Family B design (shown in Figure 4-9), with the most significant differences being the pultruded intercostals and the cobonded auxiliary frames. (The assembly tolerances are less critical for the auxiliary frames, and hence they are cobonded to the

sandwich skin at panel cure to reduce the assembly cost.) Installation of the door structure is also similar for both design families.

As for the Family B design, metal fittings are used in locations where the nature of the loading is such that complex fittings are required. This includes locations such as passenger door hinge intercostals, door sill to frame joints, and cargo floor beam splices. As for the Family B design, titanium (6Al-4V annealed plate) was originally specified in these applications to avoid the need for galvanic corrosion barrier fiberglass plies on the composite parts. However, after the initial cost estimates were generated, these elements were reconfigured to be aluminum with fiberglass isolation to realize a significant cost savings due to faster aluminum machining rates. Some representative metallic details are shown in Figure 4-10.

Assembly of the Family D Section 46 barrel proceeds in the same sequence described for the Family B design. The side panels are attached to the keel and crown panels through a mechanically fastened lap splice configuration with solid and liquid shimming used to accommodate fit-up gaps. The circumferential frames are spliced between panels with "L" sections (cut from precured braided RTM channels) using mechanical fasteners and shimming where necessary (Figure 4-24). "C"-section passenger floor beams (pultruded using IM6/8552 tape and fabric), "Z"-section passenger floor beam stanchions (hot drape formed to accommodate a joggle in the web), cargo floor beams, floor beam splices and left- and right-hand overwing longeron extensions are installed as illustrated in Figures 4-25 through 4-27. (The passenger floor beams were originally designated to use a hot drape-forming fabrication process, but pultruded elements were substituted after comparison of cost results for the Family B and Family D designs.)

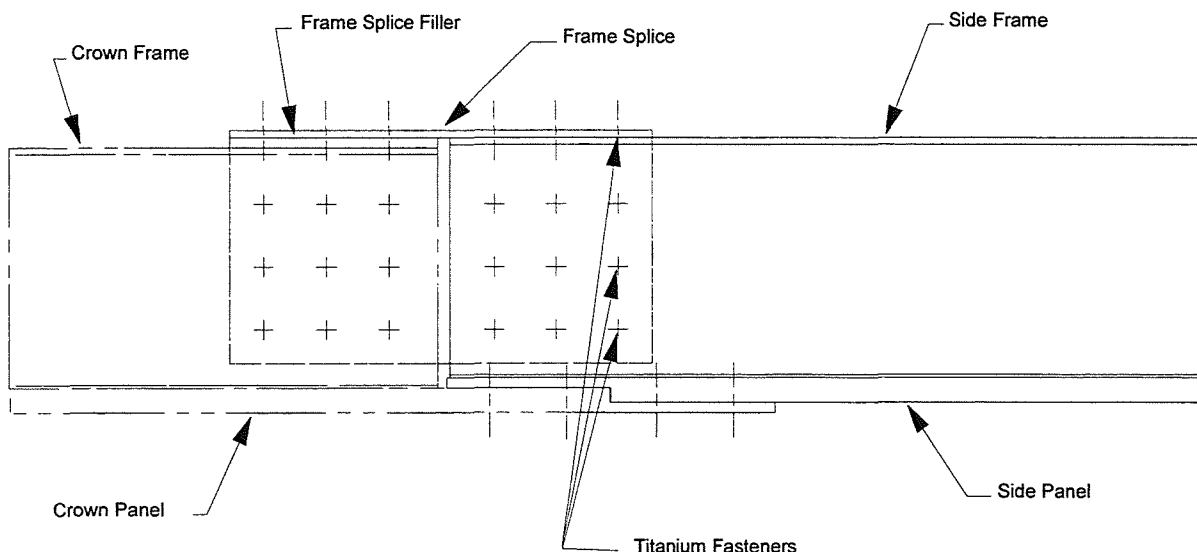


Figure 4-24. Longitudinal Splice

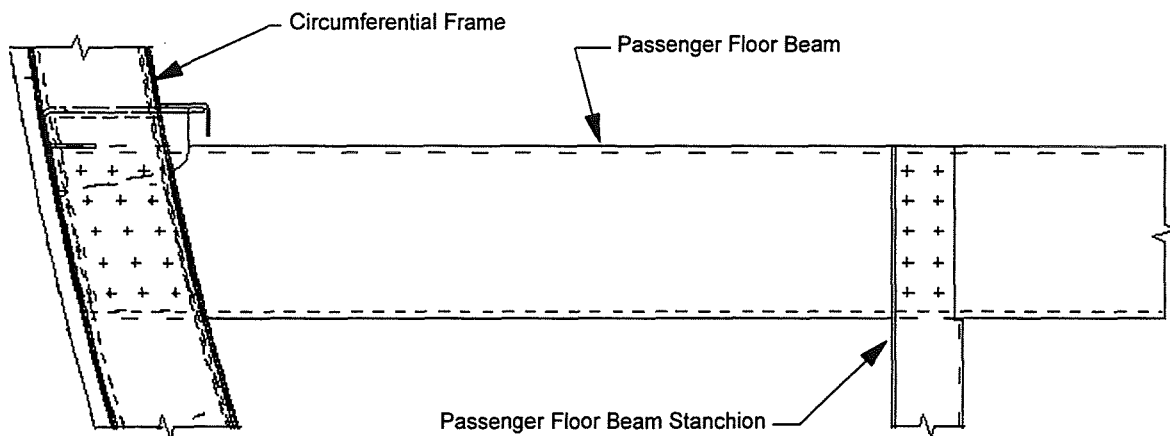


Figure 4-25. Passenger Floor Beam and Stanchion Installation

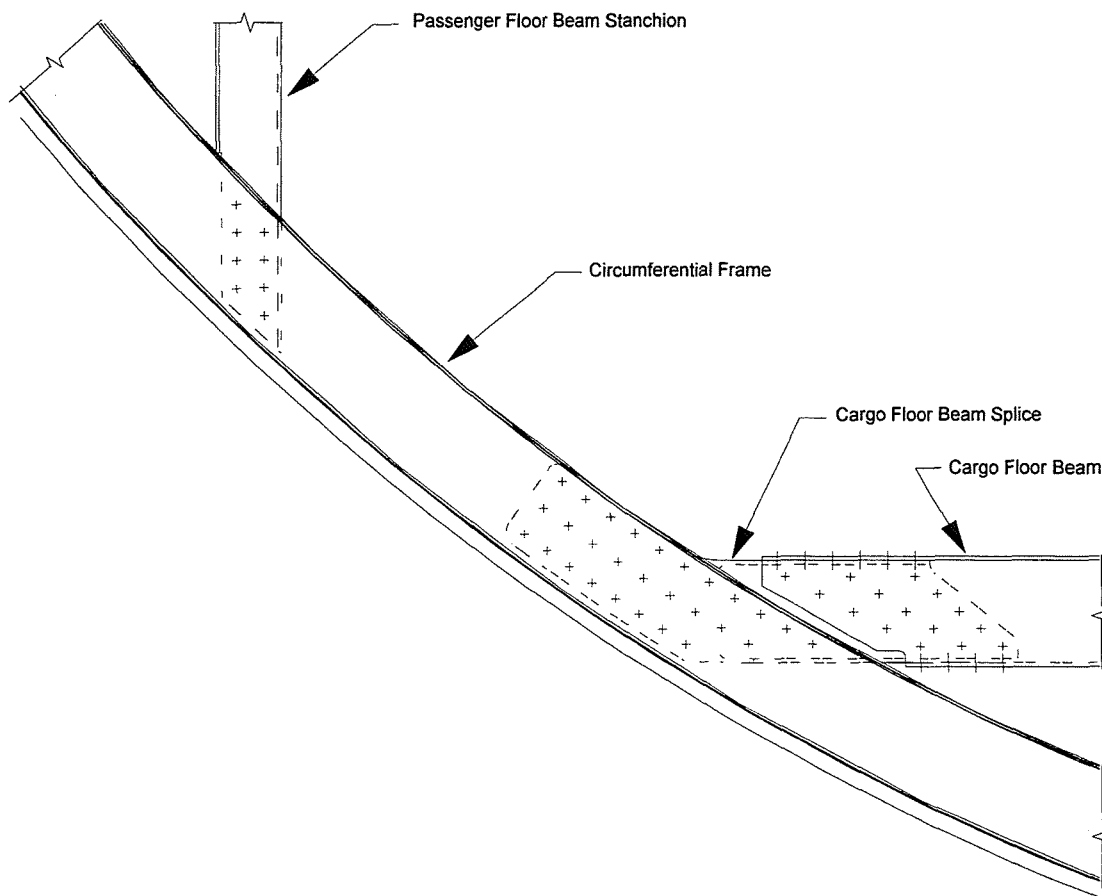


Figure 4-26. Cargo Floor Beam and Passenger Floor Stanchion Installation

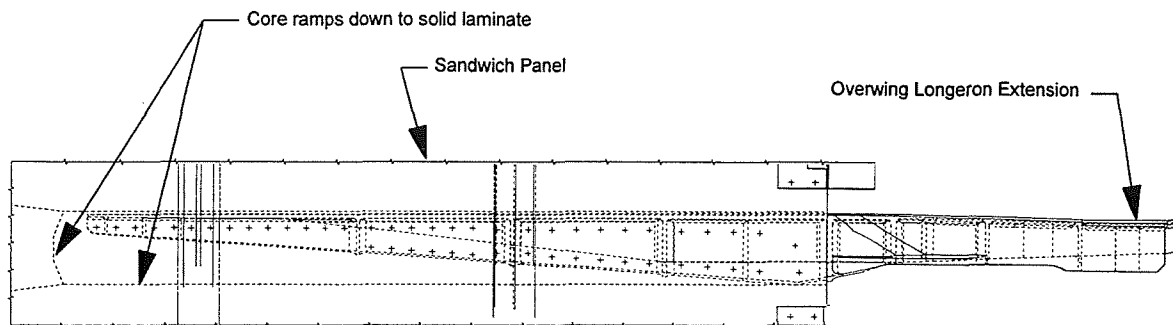


Figure 4-27. Overwing Longeron Installation

The Section 46 barrel is joined to the sections forward and aft through a butt-splice. The assembly sequence is identical to that described in Section 4.1.2 for the Family B design. Titanium Eddy-Bolts are used throughout the Family D design for mechanical assembly.

4.2.3 Family D - Critical Issues Identified

Several structural issues were identified for the Family D design as a result of the design activity. These include verification of damage tolerance performance and characterization of the sandwich configuration in the presence of stress concentrations induced by large cutouts under both static and fatigue loading. In addition, the use of full depth panel splices offers cost efficiencies over a ramped core closeout; however, the full depth splice concept needs to be verified through refined analyses and testing. There are also the historical concerns associated with sandwich structure which include repair, durability, and inspection.

Manufacturing issues for the Family D sandwich side panel are magnified by the large size of the panel, as was the case for the Family B design. Core thickness and fuselage frame tolerances will be critical to successful fabrication of the cobonded/cocured skin panel assembly. Resin transfer molding process uniformity and repeatability are concerns for the 20-foot long fuselage frames although the Family D frames are lower risk than the Family B frames due to their simplified "J" cross-section. Cost-effective layup and handling of the 600 square foot facesheet charges must be demonstrated, particularly in light of the design complexity associated with local fiberglass protection plies and provisions for systems attachments which were not explicitly addressed in this study. Further, the panel size and cocured assembly dictates that high material lay-down rates are achieved to avoid material degradation from excessive out-time. Due to the inherent stiffness of the sandwich configuration, controlling panel warpage will be critical. Also, assembly methods must be validated to ensure repeatability and reliability to maximize the efficiency of this large cost center. Finally, both production and in-service quality assurance procedures must be demonstrated. Overall, the Family D design has a higher manufacturing risk as compared to the Family B design.

4.3 Baseline Aluminum Design

To provide a comparison to the composite designs described above, an aluminum side panel design was utilized to generate cost and weight bases. The structural envelope considered was identical to that for the composite designs. Design drivers for the aluminum side panel were similar to those for the composite designs with the addition of fatigue in the regions adjacent to door and window cutouts. The panel assembly, two-thirds of which is shown in Figure 4-28, assumed state-of-the-art manufacturing techniques. Three skin panels, which feature a combination of integrally machined and mechanically fastened doublers, are spliced to form the side panel. Stringers are either machined extrusions or roll-formed J-sections. Circumferential Z-section frames were assumed to be fabricated from milled extrusion or sheet that is stretch-formed to the proper radius. The frames are connected to the skin panel with brake-formed shear ties and machined extruded stringer clips. Door structure reinforcement is assembled from milled and roll-formed sections. Some elements such as the upper and lower sills are mechanically fastened assemblies built up from individual extruded chords, sheet metal webs, and machined clips. Assembly of the panel is accomplished with a high degree of automated fastening. Furthermore, cost estimates were generated under the assumption that no manual gap gaging or hard shimming was required.

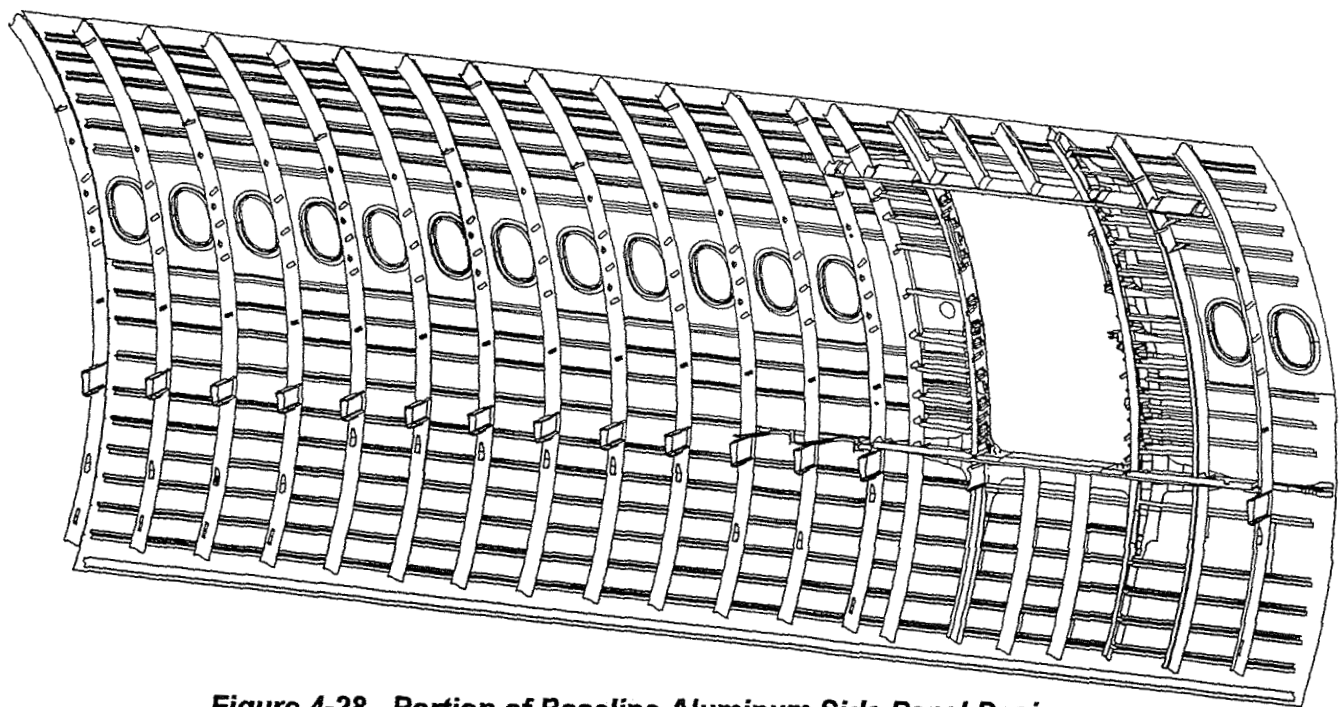


Figure 4-28. Portion of Baseline Aluminum Side Panel Design

4.4 Trade Study Results

Working directly from the engineering drawings and manufacturing plans defining the side panel components as described in the previous sections, the cost and weight of a complete side panel assembled into the fuselage barrel (excluding systems and interiors) was estimated. To improve the accuracy of the cost estimates, the manufacturing plans were broken down to the process step

level (e.g., tool clean, numerical control programming, etc.). The estimates included both nonrecurring (tooling) and recurring labor and material and followed the ground rules described in Section 3.6. Figure 4-29 depicts the trade study results normalized to the cost and weight of the baseline aluminum side panel. Further details of the composite side panel cost and weight estimates are discussed in the following two subsections.

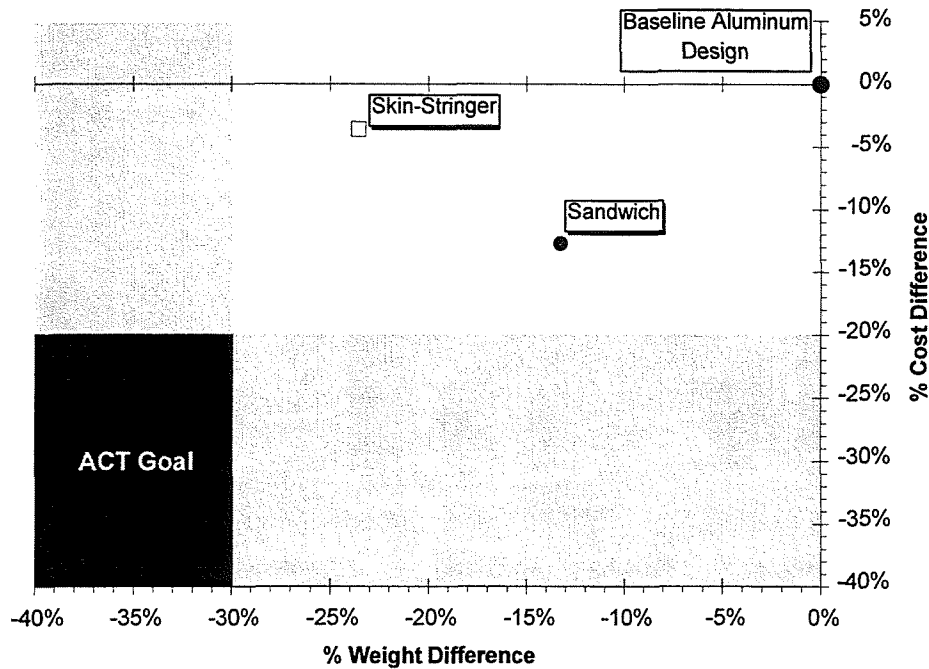


Figure 4-29. Trade Study Results Normalized to Baseline Aluminum Side Panel Cost & Weight

4.4.1 Weight Comparison

The detailed Family B (skin-stringer), Family D (sandwich), and baseline aluminum design weight estimates are summarized in Figure 4-30. The totals, depicted by the bars on the far right, indicate that the Family B design offered the greatest weight savings (23 percent savings over the metallic baseline) while the Family D design provided a 13 percent savings over the metallic baseline. For ease of discussion, the design elements comprising the side panel are grouped into nine major categories. The weight results for each of these categories are discussed below.

The single largest weight contributor to the side panel were the skins. For the skin-stringer design this included the basic skin plus doubler at the passenger door cutout while for the sandwich design the facesheets with integral doublers at the window and doors were included. The 136 pound difference between the two composite designs was attributable to (1) the lack of a window belt doubler for the skin-stringer design (it was included in the window belt weight as the doubler is integral to the window frames) and (2) nearly one-third of the sandwich panel controlled by the eight-ply facesheet minimum thickness constraint. The latter further penalized

the sandwich panel due to the design simplification to maintain identical IML and OML facesheets.

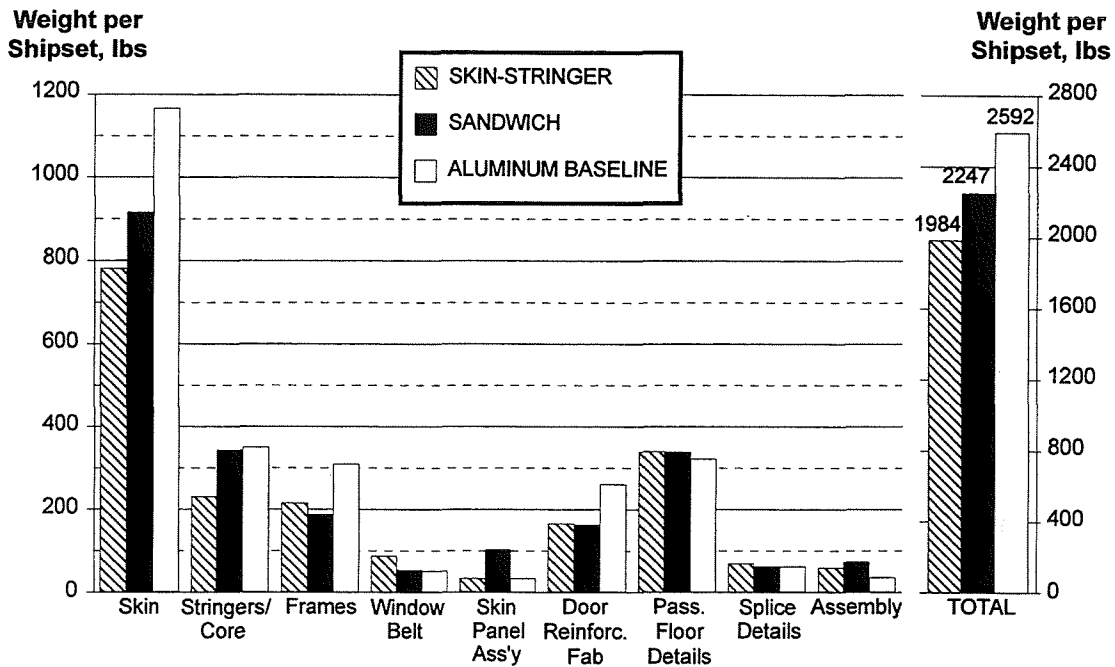


Figure 4-30. Weight Comparison of Side Panel Designs

Skin panel stiffening elements comprise the second category. The majority of the difference between the two composite designs is due to the sandwich panel core closeout elements which are responsible for over one-third (36 percent) of the core weight. The composite skin-stringer design, with wider spaced high stiffness (gained through the use of IM6 fiber) stringers, has one less stringer than the metallic panel. The graphite/epoxy density advantage over aluminum is a driver for the weight savings depicted in the fuselage frames. The differences between the two composite designs results from the mouse-holed skin-stringer frame design's larger cross-sectional area due to the required fail-safe flange and the higher number of braided plies.

Window belt weights for the composite sandwich and metallic baseline were nearly identical as both designs feature metallic frames; the sandwich design was slightly heavier due to the need for isolation between the metal frame and graphite/epoxy skin. The composite skin-stringer weight is higher than the other two due to the incorporation of skin doublers into the 3D woven window frame/doubler element developed by Lockheed.

The skin panel assembly includes all details necessary to assemble the window frames, fuselage frames, and stiffening elements to the skin. For the composite skin-stringer design this includes the weight of the adhesive (used to bond the stringers and window belt to the skin) and the fasteners (used to attach the frames to the skin). The relatively high weight for the sandwich design is primarily due to three full layers of film adhesive required to assemble the facesheets and core. Stringer and fuselage frame fasteners comprise the weight apportioned to the metallic baseline design.

skin/stringer panel floor beams, a slight weight penalty was incurred over more expensive manufacturing methods.

The splice details include the elements necessary to join the side panel to the adjoining fuselage sections (e.g., end frames, splice plates, stringer and frame splice fittings, and cargo floor beam end fittings). The composite skin-stringer and metallic baseline weights are comparable as these elements, with the exception of the end frames and splice plates, are metallic on the composite design. While the sandwich design does not have stringer splice details, the weight of the splice details is only slightly lower than for the skin-stringer design. The prime factor responsible for this is that the external splice plates are considerably thicker than the analogous elements on the skin-stringer design due to fastener countersink requirements. The sandwich panel splice requires larger diameter fasteners (with correspondingly thicker heads) to prevent excessive fastener flexibility with the full-depth splice configurations.

The final category is assembly, which includes the weight of fasteners necessary for mechanical assembly of the door structure, passenger and cargo floors, circumferential and longitudinal panel splices, and frame and stringer splices. The metallic baseline is lower than the composite designs due to the prevalent use of rivets which have little protruding volume. In spite of having no stringers to splice, the sandwich design has a higher assembly weight due to the larger diameter fasteners previously mentioned.

4.4.2 Cost Comparison

The Family B (skin-stringer), Family D (sandwich), and baseline aluminum cost results are summarized in Figure 4-31. The totals, depicted by the bars on the far right, indicate that the Family D design offered the greatest cost savings (13 percent savings over the metallic baseline compared to 4 percent savings for the Family B design). For ease of discussion, the design elements comprising the side panel are grouped into nine major categories. The cost results for each of these categories are discussed below.

The composite designs achieved the greatest cost savings relative to the aluminum baseline design in the skin fabrication. The aluminum design has significant material and labor cost components due to the large amount of machining required to fabricate these elements. The composite skins, on the other hand, benefit tremendously from the highly automated fiber placement process. The sandwich design has a slightly higher skin fabrication cost due the more expensive AS4/8552 material as compared to the AS4/938 material used in the skin-stringer design. Also, the sandwich skin cost includes window belt doubler plies; for the skin-stringer panel, these are integrated into the window frame element. The opposite situation, however, is responsible for the stiffening element cost. The baseline aluminum design utilizes automated stringer fabrication processes such as roll-forming and lightly machined extrusions whereas the composite stringers are manufactured using the drape-forming process requiring a higher degree of manual labor and more complex tooling. The sandwich design's stiffening element cost was inflated by high recurring labor costs for core closeout fabrication and manual assembly of the individual core details. The composite material cost was higher for the skin-stringer design largely due the use of IM6/8552 prepreg tape even though the core details had a higher scrap factor (1.56) than the stringers (1.45).

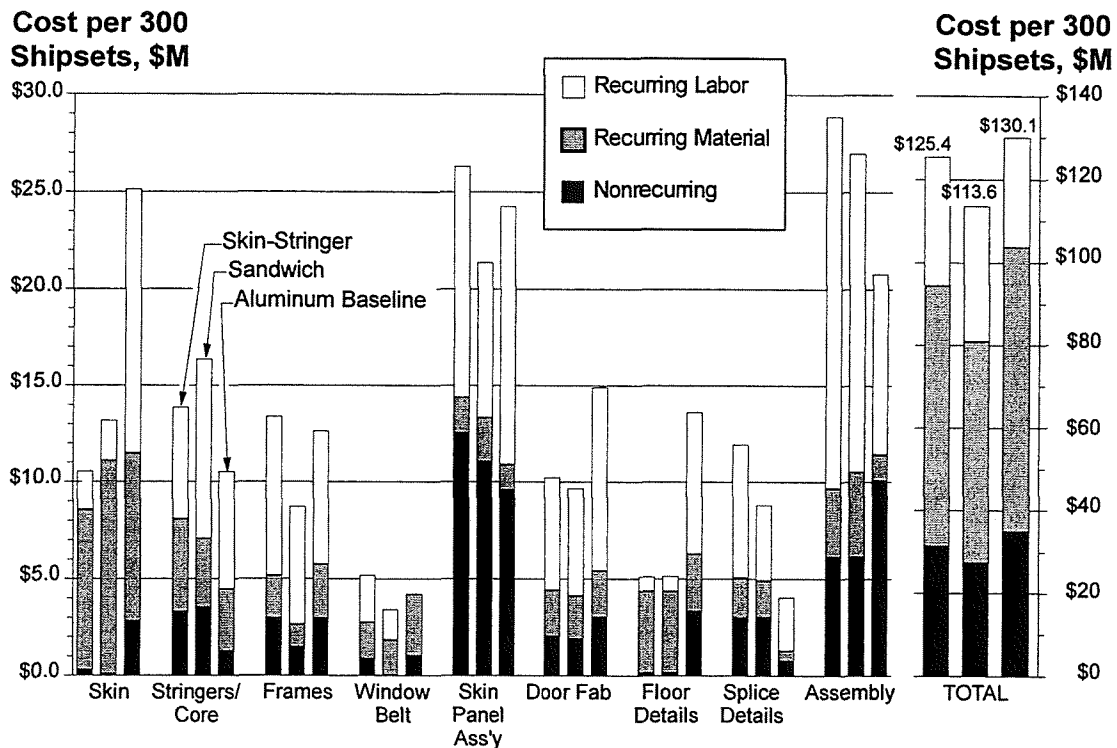


Figure 4-31. Cost Comparison of Side Panel Designs

The third design category was circumferential frame fabrication. The required fail-safe flange on the skin-stringer design accounts for the cost difference between the two composite concepts. Fabrication of the sandwich "J" frame required one braid cycle while the skin-stringer "F" frame required three braid cycles. In addition, for the "F" frame the tooling was more complicated and the material cost was higher due to higher weight and greater material scrap (a factor of 1.4 versus 1.2 for the "J" frame).

The window belt fabrication cost was similar for the three designs in spite of significant design differences. The skin-stringer design integrated the window frames, window belt skin doubler, and adjacent stringers into a single textile component. The sandwich design utilized aluminum frames isolated with fiberglass fabric from the graphite facesheets (the skin doubler was included in the skin cost).

The skin panel assembly represents a significant cost center for all three designs. For the composite skin-stringer design this cost included assembly and cobonding of the stringers to the fiber placed skin and installation of the mechanically fastened fuselage frames and stringer clips. Higher tooling, labor, and material costs associated with the fastened circumferential frames pushed the panel assembly cost higher than for the other designs. The sandwich panel assembly cost included assembly and cure/cobond of the facesheets, core blanket, and fuselage frames, and mechanically fastened assembly of the window frames. In addition, the composite designs assumed a certain level of shimming required on assembly not present in the baseline aluminum design. The baseline aluminum cost includes assembly of the three separate subpanels.

The sixth design category was the door cutout reinforcement structure fabrication. Both composite designs had very similar features and manufacturing processes and hence, virtually identical costs. Originally, the composite designs included titanium bathtub fittings but the cost associated with machining titanium was higher than the cost of aluminum fittings with isolation so the latter were substituted. The higher baseline aluminum cost reflects the high degree of tailoring in these elements. The next category, floor details, were of identical design for both composite concepts and hence there was no cost difference. Drape-formed floor beams were considered early in the design trades but were dropped in favor of pultruded beams which were less costly but slightly heavier. Similar designs were used for frame splice details in both composite designs. The skin-stringer design was more costly due to the metallic stringer splice fittings not present in the sandwich design. As for the door reinforcement details, titanium was originally considered for these but was replaced with isolated aluminum to reduce the cost.

The ninth design category was final assembly, including door structure assembly, door structure installation, side panel installation into the barrel section, and fuselage section join. Because the assembly procedures were similar for the composite designs, the tooling cost was nearly equivalent. The material cost was slightly higher for the sandwich design due to the use of a four row lap splice required by the full-depth splice configuration (the skin-stringer design featured a two row lap splice). This splice configuration also dictated the use of larger, more expensive fasteners. As for the skin panel assembly, the cost differential between the composite concepts and the baseline aluminum design is largely attributable to the assumption that assembly of the aluminum side panel was performed without shimming.

Summarizing the cost centers of the side panel global evaluation designs as a percentage of total cost for each concept results in the distribution shown in Figure 4-32. This highlights a number of interesting conclusions to be drawn from the cost assessment of the three side panel designs. First, the composite designs have similar percentages of total cost apportioned to tooling (nonrecurring) and detail part fabrication. Secondly, while the sandwich has a higher relative material cost (due primarily to the AS4/8552 facesheet material) a smaller portion of the cost is attributable to assembly as would be expected. Finally, a much larger percentage of the aluminum baseline cost results from detail part fabrication than for the composite concepts. Also, the assumption of a shimless design is reflected in the lower percentage of cost resulting from assembly.

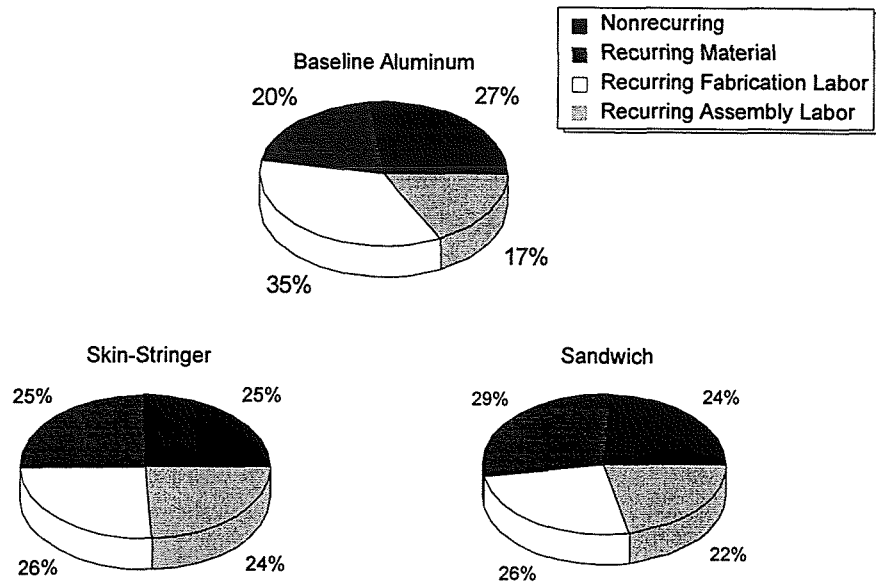


Figure 4-32. Cost Centers for the Side Panel Global Evaluation Designs

5.0 DESIGN COMPARISONS

The previous section described preliminary design development of two composite side panel concepts and estimates of the fabrication cost and structural weight for each. To fully understand the potential of each design, derivative designs were created by modifying and/or combining attractive elements of the original designs. The effect of each design variation on the total cost and weight was then estimated. An optimum design was thus developed within each family prior to down selecting to the single configuration to be further pursued during local optimization.

The following subsections describe the results of these trades, which were performed in two steps. The first step, global optimization, considered specific design detail changes to improve the design efficiency. Cost and weight estimates for the global optimization trades were performed to the same level of detail as for the original designs. The second step, local optimization potential, identified design and process improvements which the DBT felt could have a significant cost and weight impact. Estimates of the potential resulting improvements relied extensively upon past ATCAS design performance trade studies [Ref. 1, 2]. Unlike for global optimization, these relatively high risk (in terms of development required for implementation) design options were assessed using preliminary engineering analyses and cost projections.

5.1 Global Optimization

Following completion of the global evaluation cost and weight estimates described in Section 4, the DBT reviewed the data and identified several modifications intended to increase the efficiency of each design. The specific enhancements considered for the composite side panel designs are listed in Table 5-1. In this table, the percentage change from the global evaluation baseline designs is listed for each enhancement.

The Family B global optimization trades resulted in reductions of 3.7 percent cost and 2.4 percent weight from the global evaluation design. This positioned the skin-stringer design at a cost and weight savings of 7 percent and 25 percent, respectively, relative to the baseline aluminum. To reduce the cost and manufacturing risk of the Family B global evaluation design, the textile window belt element woven using powder-coated prepreg was replaced with braided/resin transfer molded (RTM) window frames as shown in Figure 5-1. The stringer cost was reduced (at a slight weight penalty) by switching from a drape-forming manufacturing process to pultrusion. The typical spacing was also increased to reduce the total number of stringers, further decreasing the stringer fabrication and circumferential splice assembly cost. And finally, the last trade reduced the panel weight by adopting a pocketed skin design with pads under the frames to preserve the required skin thickness for countersunk fasteners.

The Family D global optimization trades were more effective, particularly in regards to weight, achieving cost and weight reductions of 3.5 and 10.0 percent, respectively, from the global evaluation design. The braided/RTM window frames utilized on the B1 design were adopted to eliminate costs associated with galvanic isolation of the skin panel from the aluminum frames. A new, lighter (albeit more costly) core material was employed to reduce weight. Both cost and

weight reductions were possible through improvement of the core close-out details. And lastly, a more efficient cure cycle was adopted which reduced the amount of tooling and labor required to assemble the sandwich skin panel. These design improvements placed the Family D design at cost and weight reductions of 16 percent and 22 percent, respectively, compared to the baseline aluminum.

	Element	New Feature	Replacing . . .	Cost Change	Weight Change
Skin-Stringer	1) Window Belt	Lockheed 2D braided textile frame w/ AFP skin doubler	Lockheed 3D woven window belt	-0.5%	-1.1%
	2) Stringers	reduce number of different stringers - constant cross-section & layup - pultruded	11 unique stringer part numbers - drape-formed	-1.9%	+0.6%
	3) Stringers	9.1" spacing (eliminates one stringer)	8.4" spacing (lower side)	-1.1%	-0.9%
	4) Skin	discrete pads under frame attach flanges to provide required thickness for countersunk fasteners	overall min. skin gage based on countersink requirements (15 plies min.)	-0.2%	-1.0%
Sandwich	1) Window Frame	Lockheed 2D braided textile frame	aluminum frame w/ fiberglass isolation	-2.5%	-1.4%
	2) Core	4.5 pcf Korex	8.0 pcf HRP	+1.6%	-3.6%
	3) Core Close-outs: Window Belt Circ. Splice Door	5° core ramp to solid laminate @ window belt 20° core ramp to solid laminate @ circumferential splice solid laminate under intercostals only	injection molded Gr/Ep ring close-out solid laminate core close-out solid laminate between edge & aux. frames	-1.5%	-3.6%
	4) Skin Assembly	eliminate one ply of adhesive with 2-stage panel bond process - cocure OML facesheet & core splices - cocure IML facesheet to core/OML facesheet bond ass'y	panel bond process - cocure one ply of fabric to core assy	-1.1%	-1.4%

Table 5-1. Side Panel Global Optimization Trades

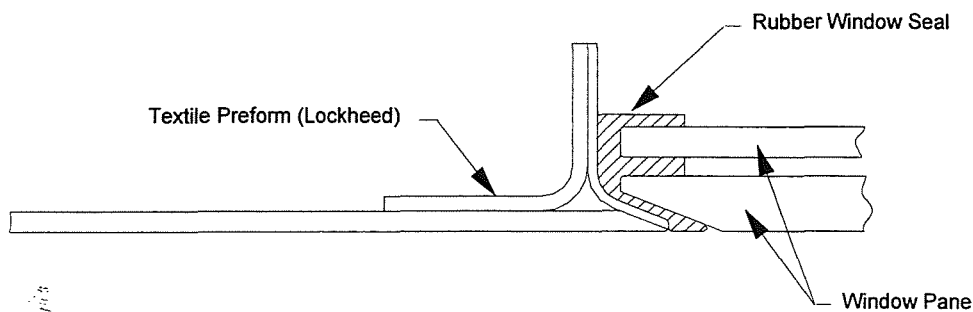


Figure 5-1. Braided RTM Individual Window Frame Element

Figure 5-2 summarizes the relative cost advantages of the three concepts compared in this trade study. The Family B design offers cost benefits over the baseline aluminum in the areas of skin, door reinforcement structure, and floor detail fabrication. In addition to those areas, the Family D design has lower projected costs for circumferential frame fabrication and panel subassembly. The composite designs were not able to achieve cost savings in the areas of stringer and core fabrication, driven by complex tooling, material costs, and a higher proportion of manual labor, and panel installation, due to the shim-free assembly assumption for the baseline aluminum panel.

Manufacturing Step	Cost Advantage			Comments
	Skin-Stringer	Sandwich	Baseline Aluminum	
Skin Fab	✓	✓		Lower composite skin weight and automated layup of single large panel.
Stringer/Core Fab			✓	Complex tooling & higher material cost for composite designs.
Frame Fab		✓		Sandwich concept affords simple J frame configuration.
Panel Subass'y		✓		Reduced labor and part count for sandwich concept.
Door Fab	✓	✓		Composite concepts utilize efficient fab processes. Further design refinement may erode cost savings.
Floor Details	✓	✓		Composite design features efficient pultrusion process.
Panel Install.			✓	Aluminum baseline reflects shim-free design and lower fastener cost.

Figure 5-2. Comparison of Cost Advantages Offered By Composite Designs

5.2 Local Optimization Potential

Building upon the results of the global optimization trades, the DBT identified potential design modifications to further improve the cost and weight efficiency of the composite concepts through review of dominant cost drivers. These changes were important in answering "what if?" questions regarding the influence of selected design details even though the changes were typically accompanied by increased risk. In addition, this exercise highlighted structures and manufacturing issues to be resolved during subsequent local optimization of the selected design concept. Table 5-2 lists, for each concept as a percentage change from the composite global evaluation design, the individual contribution of each trade considered.

For the Family B design, assembly was a major cost center. To assess the impact of assumptions made regarding the required level of shimming, estimates reflecting shimless assembly were generated. Based on these estimates it was found that such a design would reduce the Family B cost by nearly 14 percent. The second trade drew upon the ATCAS crown studies to determine the influence of widely spaced hat stringers in lieu of the global evaluation design's more closely spaced "J" stringers. This was found to have only a slight benefit, due in part to the stringer spacing constraints imposed by the overall side panel configuration (e.g., passenger door and floor locations) and the higher compression-shear loading in the lower side than was present in the crown. The cumulative result of these trades indicated that the Family B design had the

potential to be 23 percent less costly and 23 percent lower weight relative to the baseline aluminum.

	Element	New Feature	Replacing . . .	Cost Change	Weight Change
Skin-Stringer	1) Assembly	eliminate gap gaging and shim assumptions	all assemblies gaged and 15% of joint area requiring hard shims	-13.9%	-
	2) Stringers	hat stringers at ~13" spacing	J stringers @ ~8" spacing	-5.0%	+3.4%
Sandwich	1) Assembly	eliminate all gap gaging and shimming	all assemblies gaged and 15% of joint area requires hard shims	-9.1%	-
	2) Long. Splice	2 rows of fasteners	4 rows of fasteners	-1.6%	-0.5%
	3) Skins	eliminate one ply per facesheet (less conservative DT criteria)	DT criteria based on ATCAS database	-1.5%	-3.6%
	4) Frames	42" frame spacing	21" frame spacing	-13.4%	-1.5%
	5) Intercostals	eliminate door stop intercostals	16 door stop intercostals	-2.3%	-0.6%
	6) 360° Barrel	eliminate panel longitudinal splices	4 longitudinal splices	-4.9%	-2.7%

Table 5-2. Side Panel Local Optimization Potential Trades

Assembly was also a major cost center for the Family D design, so a similar trade was performed to assess the effect of eliminating all shimming on assembly. This provided significant cost savings, although not as substantial as for the Family B design due to the smaller amount of assembly in the sandwich design. To further reduce assembly cost and weight, the longitudinal splices were reconfigured from four fastener rows to two fastener rows. The effect of several additional trades were also considered, including (1) adopting a less conservative damage tolerance criteria which allowed one ply per facesheet to be eliminated, (2) increasing fuselage frame spacing, and (3) eliminating door stop intercostals, all of which took advantage of inherent characteristics of sandwich construction. Of these, increased frame spacing provided the most significant cost reduction (13.4 percent) while the less conservative damage tolerance offered the greatest weight savings (3.6 percent). Finally, to assess the influence of panel splices on the side panel cost and weight, a 360° sandwich barrel concept was estimated. This eliminated all details associated with skin panel longitudinal splices, reducing the cost and weight by an additional 4.9 percent and 2.7 percent, respectively. Summing the contribution from each of these trades indicated that the sandwich design potentially offered 40 percent cost savings and a 29 percent weight reduction relative to the baseline aluminum.

The results of both the global optimization and local optimization potential trades are depicted in Figure 5-3. This plot shows the total projected cost and weight savings for both composite designs considered during the side panel global evaluation effort relative to the baseline aluminum. Note that the ACT program goals reflect the additional benefit of resizing the overall aircraft to take advantage of lower structural weight; this benefit was not included in the Family B or Family D side panel cost and weight estimates.

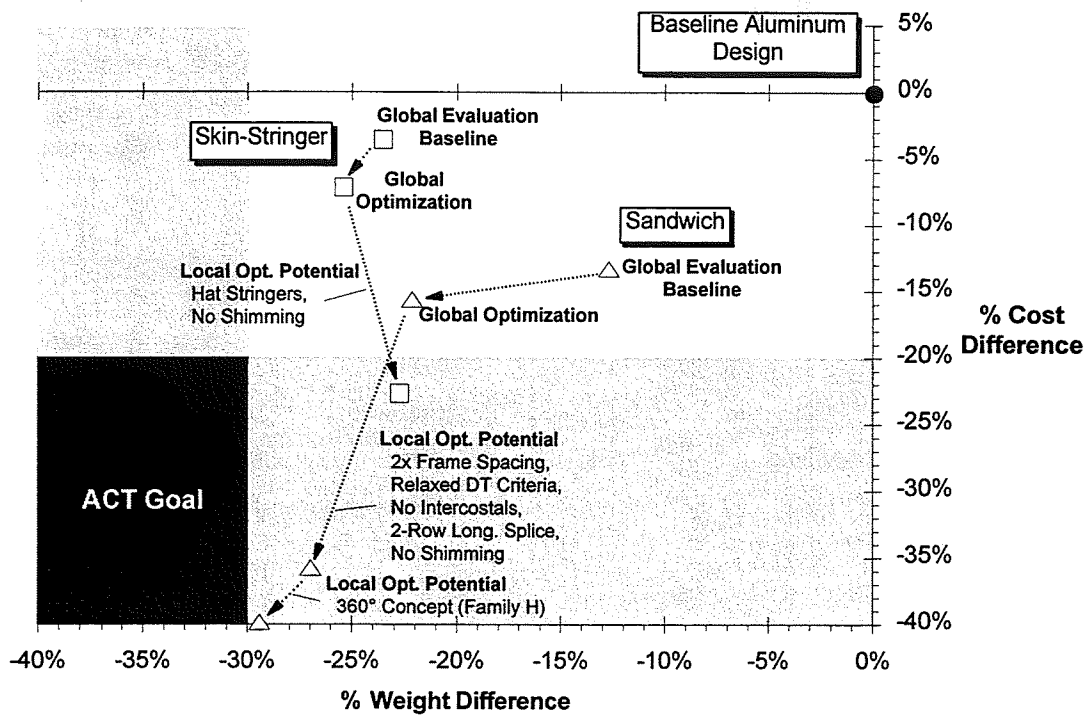


Figure 5-3. Results of Side Panel Global Optimization Trades

5.3 Design Trades Assessment

The trades studies described in the preceding section quantified the potential weight and cost savings the two composite side panel design configurations offer over the baseline aluminum. Due to the varying levels of risk introduced with each design variation, the viability of the composite design concepts was assessed from structural performance and assumed manufacturing process viewpoints. This assessment was performed in two steps: (1) determining the probability of attaining the full cost and weight savings potential of each design concept through the use of innovative design features and (2) quantifying the cost impact if assumptions regarding design details, structural performance, and manufacturing processes are not realized. These risk assessments also enabled prioritization of the manufacturing methods, materials, and design details to be pursued for maximum cost and weight benefit during subsequent side panel local design optimization.

5.3.1 Probability of Attaining Projected Cost/Weight Savings

As described in Sections 5.1 and 5.2, further cost and weight reductions may be realized through implementation of alternate manufacturing processes, materials, and innovative design details. A risk assessment was conducted to understand the probability of achieving the benefits resulting from these design changes. This evaluation (a) quantified risk the DBT attributed to achieving the cost/weight potential of each design and process improvement identified in the global optimization and local optimization potential trades, and (b) identified local optimization development priorities. Two probabilities, one based on meeting structural requirements and the other based on manufacturing feasibility, were projected for each design improvement. A

combined probability was then obtained by multiplying the two component probabilities as shown in Table 5-3. A probability ranking of 100 percent indicates that the DBT foresaw zero risk in meeting the manufacturing and structural performance criteria associated with that particular design feature (relative to the global evaluation baseline designs). The results of this assessment are plotted in Figure 5-4 relative to the baseline aluminum.

	Design Enhancement	Probability of Success		
		Manufacturing Feasibility	Structural Performance	Combined
Skin-Stringer	Braided RTM window frame	90%	90%	81%
	Pultruded stringers	60%	90%	54%
	Increased stringer spacing	100%	95%	95%
	Pocketed skin	100%	70%	70%
	Improved Assembly			
	Liquid shims on assembly only	80%	100%	80%
	No gap gaging or shimming on assembly	30%	100%	30%
	Hat stringers	80%	100%	80%
Sandwich	Braided RTM window frame	90%	90%	81%
	Korex honeycomb core	100%	80%	80%
	Improved core closeouts	60%	70%	42%
	Enhanced panel bond process	50%	90%	45%
	Improved Assembly			
	Liquid shims on assembly only	70%	100%	70%
	No gap gaging or shimming on assembly	30%	100%	30%
	Two-row longitudinal splice	90%	70%	63%
	Reduced minimum gage	100%	60%	60%
	Increased frame spacing	100%	90%	90%
	Eliminate door-stop intercostals	100%	30%	30%

Table 5-3. Probabilities of Success for Side Panel Design Improvements

For the composite skin-stringer design, the DBT attributed a relatively high probability of success to implementing a weight efficient "pocketed" skin, hat stringers, and part tolerances within ± 0.030 inch to eliminate the need for hard shims on assembly. These design improvements accounted for roughly half the cost reduction and virtually all the weight savings over the baseline design. Much lower probabilities of success were attributed to pultruded stringers and the ability to totally eliminate all forms of shimming on assembly. The primary concerns with these design features were concerns over cobonding pultruded stringers (due to potential for die lubricants to contaminate the resin system) and the ability to accurately control part dimensions to the high tolerances (± 0.006 inch) required to allow shimless assembly.

Similarly, for the composite sandwich design, the DBT attributed high probabilities of success to increased frame spacing, Korex honeycomb, and liquid shimmed assemblies. Lower probabilities were assigned to a simplified longitudinal splice, a lighter circumferential splice, improved sandwich close-outs, and a damage tolerance criteria less conservative than that used

for the global evaluation design. Considering only the design changes with probabilities greater than 50 percent still would place the sandwich design at a desirable cost/weight position relative to the skin-stringer design at a similar probability level. Those design features falling below 50 percent included elimination of door intercostals, no shimming on assembly, and improved window belt sandwich core close-outs. The low probability assigned to these design features was due to lack of detail generated during global evaluation and the challenge of controlling part dimensions to the high tolerances required for shimless assembly.

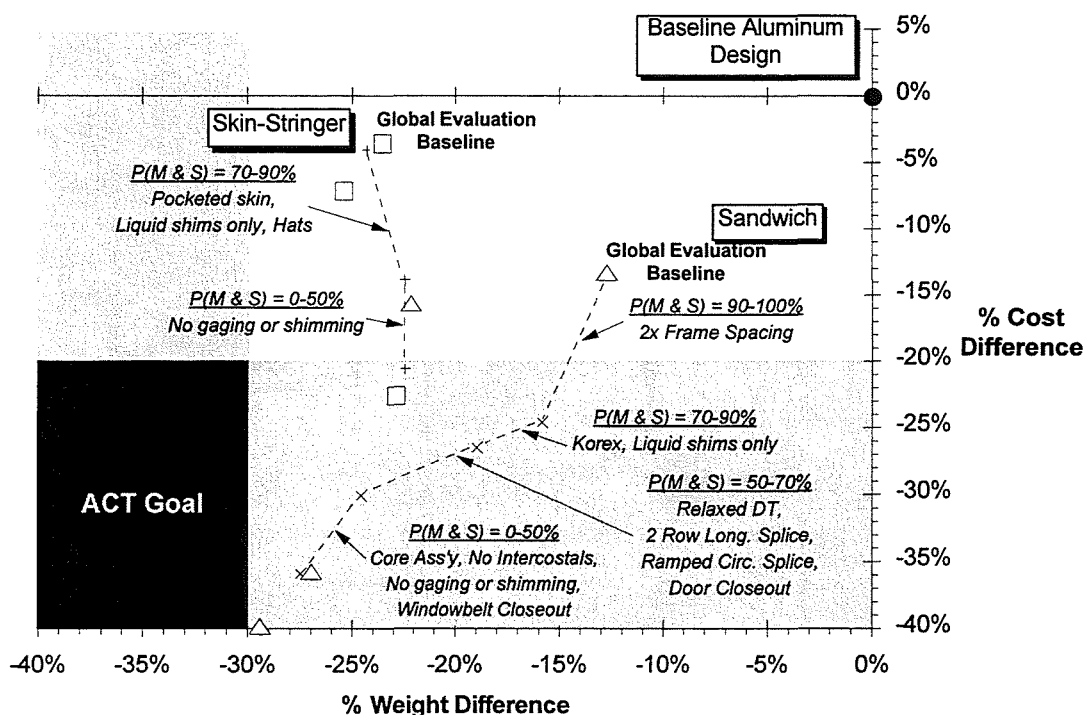


Figure 5-4. Probability Assessment of Achieving Cost/Weight Projections Identified During Local Optimization Potential

5.3.2 Cost Estimate Sensitivity Analysis

Balancing the outlook for further efficiency improvements of the composite designs was quantification of the sensitivity to assumptions upon which the cost estimates were predicated. Figure 5-5 shows the most significant cost impacts (relative to the aluminum baseline design) if projected trade study manufacturing and design assumptions cannot be met. (Recall that the baseline aluminum design assumes high tolerance automated manufacturing methods requiring no shimming on assembly.) This analysis confirmed that all design points identified in the study would be equally sensitive to the cost assumptions (i.e., all would increase by similar amounts). If material prices based on current market conditions were used in lieu of those based on projected future large-quantity production orders, the total side panel costs would increase by about 8 percent. If bonded or cocured barriers used to prevent galvanic corrosion of aluminum fittings were too costly or were deemed inadequate, reverting to machined titanium fittings would add 6 percent to the cost. Automated fiber placement rates remaining at developmental rates of 5 lbs/hr (instead of 25 lbs/hr as assumed in the estimates) increased the total cost by

3 percent. An increase in automated tow placement process material scrap rates from 10 percent to 50 percent would escalate the cost an additional 7 percent. The skin-stringer and sandwich designs would both see a cost increase of 1 percent for tailoring fuselage frames to provide greater weight savings in the face of varying loads around the circumference; decreased panel bond assembly efficiency would have a similar impact for both concepts. And finally, significant cost escalation would result if the overall factory efficiency of all processes was lower than expected, again effecting both concepts by a comparable magnitude as the skin-stringer and sandwich designs had similar total labor-hour estimates.

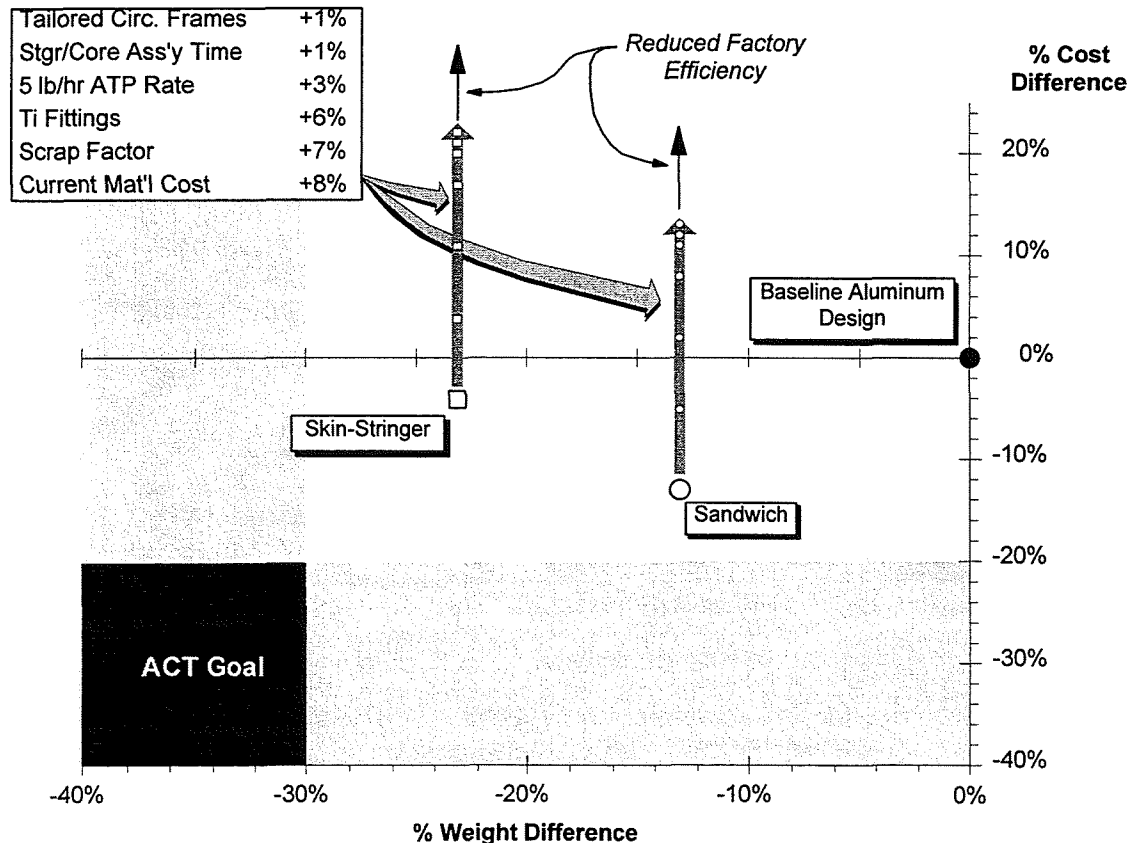


Figure 5-5. Effect on Total Cost if Assumptions Are Not Realized

5.3.3 Total Concept Cost/Weight Potential

With completion of this risk assessment, the DBT was able to quantify each concept's potential to meet the ACT program goals. This is depicted in Figure 5-6 with the region defined for each design representing the range of cost and weight savings identified during the side panel global evaluation trade study. This region includes both the potential related to concept optimization (variations in design detail and manufacturing processes) described in Sections 5.1 and 5.2 as well as savings erosion if key assumptions were not realized as discussed above.

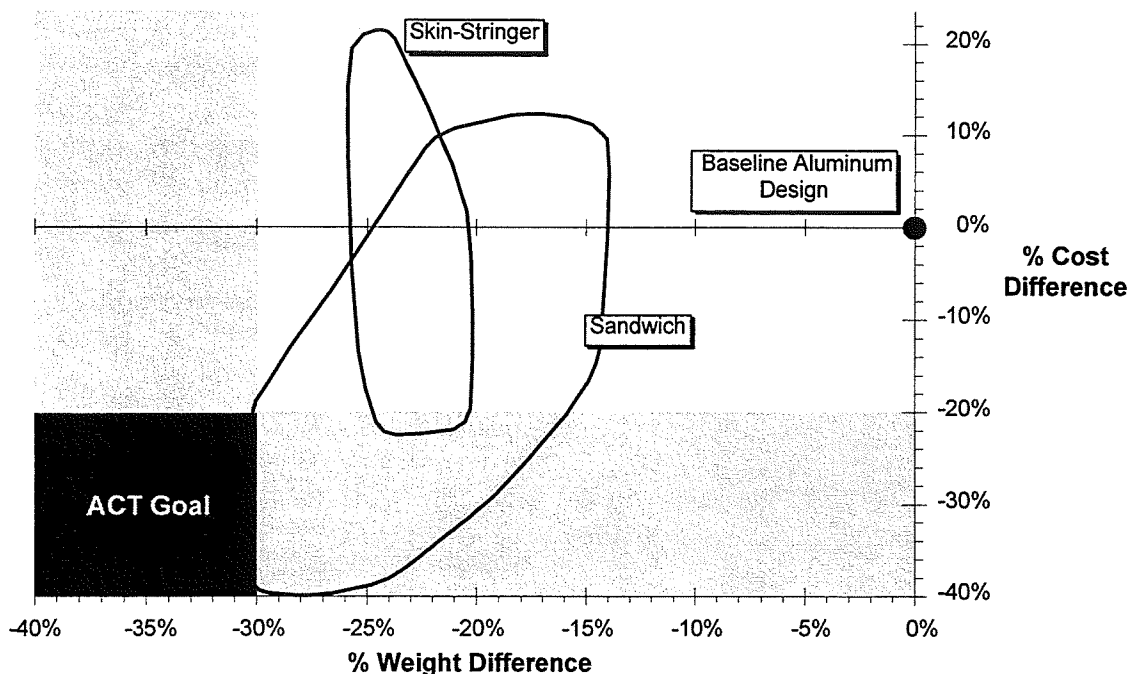


Figure 5-6. Cost & Weight Space Studied for Composite Skin-Stringer and Sandwich Concepts

5.4 Concept Selection Rationale

This section summarizes the process the side global DBT followed to select the preferred design concept. Section 5.5 prioritizes manufacturing and technical issues for the selected concept and presents future plans to mitigate these risks through remaining Phase B efforts.

The ATCAS program performs global evaluation to select a design family with cost and weight savings potential prior to the focused manufacturing and structures scaling efforts in subsequent stages of development. Identification of the manufacturing and technical risks for a selected concept are also important results from global studies because they help prioritize development of critical details.

5.4.1 Synopsis of Data Available for DBT Decision

Global evaluation has typically been a laborious DBT effort because cost and weight estimates are desired for concepts with representative structural design and manufacturing detail. Side global evaluation required significantly more time and resources than the crown or keel trade studies. Characteristics of the side panel leading to this increased effort include larger panel size, multiple critical load cases, and additional design details (e.g., passenger floor structure, and passenger door and window cutout reinforcements).

As shown in Figure 5-7, side global studies started in 1992 with DBT selection of key design features for the composite skin-stringer and sandwich concepts. Design sizing, engineering drawings, and manufacturing pre-plans required over a year to complete. Throughout this time frame, numerous team interactions matured each design based on insights for performance,

producibility, and cost and weight savings. As discussed in Sections 5.2 and 5.3, the trade study results were supplemented with evaluation of additional local optimization potential and manufacturing risk contributing to differences between the two concepts. Although some attempt was made to quantify the potential and risk of global crown and keel concepts (Refs. 1 and 2), a more rigorous study was performed for the side panel to evaluate the significance of higher costs for the composite skin-stringer design. Documentation of the ground rules, technical assumptions, major DBT decisions, and results occurred throughout side global evaluation, facilitating a complete review of the trade study prior to concept selection.

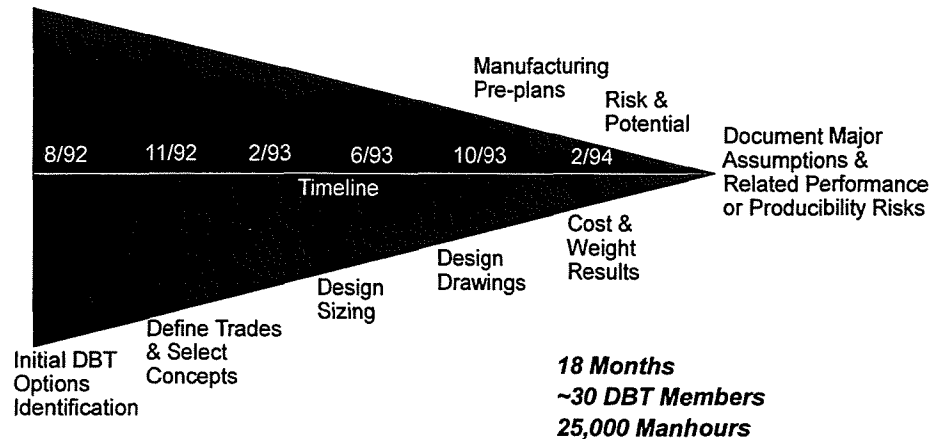


Figure 5-7. Side Global Evaluation Timeline

5.4.2 Team Concept Selection

Information contained within this contractor report represents a summary of the side global database. The data in this report was available at the time of concept selection in the form of a notebook for DBT review. Twenty-six DBT members met for two days to review the global evaluation data, discuss results, and select the side panel concept to be pursued during local optimization. These meetings included team members from Boeing (18), Lockheed (4), Northrop (3), and Sikorsky (1). The agenda for these meetings included presentations and discussions on:

- design and manufacturing details for each concept,
- cost estimating methods,
- baseline aluminum cost and weight,
- composite concept cost and weight results,
- potential and risk analyses,
- a method to quantitatively evaluate both design concepts,
- concept selection, and
- future plans.

The concept preference form shown in Figure 5-8 was completed by each DBT member attending the final side global DBT meetings to select the preferred concept and provide narrative rationale. Following review of technical issues and the side global trade study cost and weight results, each team member completed the selection form. The DBT selection process

resulted with eighteen members in favor of the sandwich concept and five proponents of the skin-stringer concept. A preference for the sandwich structure was primarily due to the trade study results indicating it had the greatest potential for meeting NASA ACT program cost and weight savings goals versus state-of-the-art aluminum structure.

1/27/94

Team Member

Name

Affiliation

Check One

☐

Composite Skin/Stringer (Family B)

Pursue Unique Local Optimization Potential: Yes No

If Yes, Please Discuss Below

☐

Composite Sandwich (Family D)

Pursue Unique Local Optimization Potential: Yes No

If Yes, Please Discuss Below

Comments Supporting Preferred Concept

Influencing Factor	Brief Description
Credibility of Cost Results	
Value of Weight Savings	
Cost Saving Potential	
Weight Saving Potential	
Other	

Figure 5-8. Team Member Input Form Used to Collect Preferred Concept and Supporting Rationale

In addition to selecting the preferred side panel concept, each DBT member was requested to comment on factors influencing the selection. These influencing factors included:

- 1) Credibility of the cost estimate results - were the cost estimates deemed accurate, and were the estimates for each concept performed to the same fidelity?
- 2) Value of weight savings, e.g., a high value for weight saved would essentially capture some of the life cycle cost savings that are important to the airlines,
- 3) Cost saving potential for each concept,
- 4) Weight saving potential for each concept, and
- 5) Any other factor(s) that influenced concept selection, i.e., manufacturing and performance risk, and the perceived durability and repairability of each concept.

Results from this influencing factor survey are presented in Table 5-4 showing the number of DBT responses for each factor. The results were sorted into three qualitative categories (high, average, and low) for each factor. Additional comments were summarized and have been included in Table 5-4. The DBT considered that the cost estimates were performed fairly and with reasonable accuracy, and typically allocated an average or high value for weight saved. The

consensus was that the sandwich design had greater cost saving potential and provided the best opportunity to reach the ACT cost-savings goal. The DBT also thought that the sandwich design had better weight saving potential, despite the fact that the global evaluation results (prior to optimization trades) indicated that the skin-stringer concept was more weight-efficient. The DBT, while selecting the sandwich design for continued development due to the better cost savings potential and the potential to meet the ACT goals, was concerned about performance, manufacturing, durability, and repair risks.

Influencing Factor	SKIN/STRINGER			SANDWICH			Typical DBT Comments
	High	Average	Low	High	Average	Low	
Cost Savings Credibility	0	15	2	0	15	2	Average cost credibility for both concepts
Weight Savings Value	4	9	0	5	9	0	Average to good weight value for both
Cost Savings Potential	1	2	4	16	2	0	Much better cost savings potential for sandwich
Weight Savings Potential	7	2	4	12	3	0	Better weight savings potential for sandwich
Performance Risk	0	0	0	3	1	0	More risk perceived for sandwich
Manufacturing Risk	0	0	0	2	2	2	More manufacturing risk for sandwich
Durability & Repair Risk	0	0	2	7	0	0	More repair and durability risk for sandwich
Potential for Reaching ACT Goals	5	0	1	12	1	3	ACT goal potential better for sandwich

Table 5-4. Side Panel Concept Selection - DBT Members Comments

The selection of a sandwich design superseded the original Family C side panel baseline concept (Ref. 1). Trade study results and team arguments justifying the selection of sandwich were compiled for technical review by Boeing and NASA management. Approval for more detailed studies to develop composite sandwich fuselage technology through the remainder of Phase B was obtained in these reviews. A prioritized list of sandwich manufacturing and structural performance issues to focus near-term ATCAS efforts were also identified during the course of these meetings. Section 5.5 discusses these issues as well as plans to demonstrate the suitability of composite sandwich structures for transport fuselage applications.

5.4.3 Team Evaluation of Global Cost & Weight Trades

In addition to selecting their preferred concept, most DBT members (20 of 26) filled out quantitative evaluation forms. These forms, similar to the samples shown in Figure 5-9, were used to quantify team member assessments of the side global cost and weight trades. Team members were requested to quantify the combined effects of concept potential and risks on achievable cost and weight. Trade study results compiled for the DBT review itemized cost and

weight differences due to specific design features and assumptions, allowing judgments on the risk and potential of specific changes. Note that each team member evaluated both skin-stringer and sandwich concepts, regardless of individual preference.

Side Quadrant Global Cost & Weight Trade Study Concept Evaluation Form for: Composite Skin/Stringer (Family B)		1/27/94
		Team Member
		Name
		Affiliation
Evaluation of Global Study for Family B Composite Skin/Stringer Design Concept	1) Relative Cost (vs. Aluminum Baseline)	0.958
	2) Relative Cost from not Achieving Mfg. Advancements (0 to 0.244)	
	3) Projected Value of Weight Savings (\$0 to \$250 per lb of weight savings)	\$/lb
	4) Weight Saved (Aluminum Baseline - Skin/Stringer)	630 lb
	5) Baseline Aluminum Average Cost per Shipset	\$433,643
	6) Relative Value of Weight Saved (Line 3 x Line 4 ÷ Line 5)	
	7) Subtotal Assessment of Skin/Stringer Global Concept (Line 1 + Line 2 - Line 6)	
Evaluation of Global Optimization and Local Opt. Potential of Family B Composite Skin/Stringer Design Concept	8) Additional Weight Savings Potential (0 to 65 lb)	
	9) Additional Relative Cost Savings Potential (0 to 0.195)	
	10) Relative Value from Additional Weight Savings Potential (Line 3 x Line 8 ÷ Line 5)	
	11) Subtotal Assessment of Skin/Stringer Concept Potential (Line 9 + Line 10)	
	12) Total Assessment of Composite Skin/Stringer Concept (Line 7 - Line 11)	

Side Quadrant Global Cost & Weight Trade Study Concept Evaluation Form for: Composite Sandwich (Family D)		1/27/94
		Team Member
		Name
		Affiliation
Evaluation of Global Study for Family D Composite Sandwich Design Concept	1) Relative Cost (vs. Aluminum Baseline)	0.851
	2) Relative Cost from not Achieving Mfg. Advancements (0 to 0.244)	
	3) Projected Value of Weight Savings (\$0 to \$250 per lb of weight savings)	\$/lb
	4) Weight Saved (Aluminum Baseline - Sandwich)	377 lb
	5) Baseline Aluminum Average Cost per Shipset	\$433,643
	6) Relative Value of Weight Saved (Line 3 x Line 4 ÷ Line 5)	
	7) Subtotal Assessment of Sandwich Global Concept (Line 1 + Line 2 - Line 6)	
Evaluation of Global Optimization and Local Opt. Potential of Family D Composite Sandwich Design Concept	8) Additional Weight Savings Potential (0 to 320 lb)	
	9) Additional Relative Cost Savings Potential (0 to 0.195)	
	10) Relative Value from Additional Weight Savings Potential (Line 3 x Line 8 ÷ Line 5)	
	11) Subtotal Assessment of Sandwich Concept Potential (Line 9 + Line 10)	
	12) Total Assessment of Composite Sandwich Concept (Line 7 - Line 11)	

Figure 5-9. Team Member Trade Study Quantitative Evaluation Forms

The quantitative forms for each concept calculated projected deviations from the global evaluation baseline and required input on:

- i. additional cost resulting from not achieving manufacturing advancements (a risk assessment)
- ii. cost savings resulting from implementation of global optimization and local optimization potential design details
- iii. weight savings resulting from implementation of global optimization and local optimization potential design details
- iv. value of weight savings (i.e., admissible cost increase for decreased weight in the allowable range of \$0/lb to \$250/lb).

Team members chose from weight savings identified in the global optimization and local optimization potential studies (i.e., values greater than or equal to the global evaluation baseline). Team members selected the achievable cost for each composite concept within a range from savings identified in optimization studies to possible cost increases quantified through risk assessment. Composite cost and weight inputs (items i, ii, and iii) were normalized as a percentage of the baseline aluminum. Composite weight saved versus the baseline aluminum was also converted to a relative cost through the parameter quantifying the value of weight savings (item iv from above). A total relative cost was calculated by summing terms for all four components.

Results for the Family B design are shown in Figure 5-10 (not including perceived value of weight savings), plotted as a function of the percent difference relative to the baseline aluminum. Averaged cost and weight savings projected by the team are listed with associated standard deviations. Figure 5-10 indicates team members believed that weight savings identified in optimization phases of the trade study were achievable. The standard deviation for weight results is indicative of the small additional weight savings identified in quantifying skin-stringer optimization potential. The assessment of cost showed team members believed that the potential and risk would tend to balance, yielding an achievable cost savings close to that of the global evaluation baseline. Note that the team assessment of the skin-stringer concept places it nearly 20 percent away from the ACT cost goals.

Results for the sandwich concept are shown in Figure 5-11 (not including perceived value of weight savings), again plotted relative to the aluminum baseline. This figure indicates team members believed that a considerable portion of the weight savings identified through optimization phases of the trade study were achievable. The team average weight savings represents a large shift in sandwich weight savings from the global evaluation baseline point towards that projected for Family B. The larger standard deviation for sandwich weight results is indicative of the additional weight savings identified in quantifying sandwich optimization potential.

As was the case for Family B, team members believed that cost potential and risk of the sandwich concept would tend to balance, yielding an achievable cost savings close to that of the global evaluation baseline. For the sandwich concept, team assessment of achievable cost savings was much closer to ACT program goals than for the Family B design. Although standard deviations for team member inputs on achievable cost savings for both concepts were

relatively large, a higher value was obtained for the Family D design. This is likely attributable to the greater risks DBT members associated with achieving additional cost savings of the sandwich concept.

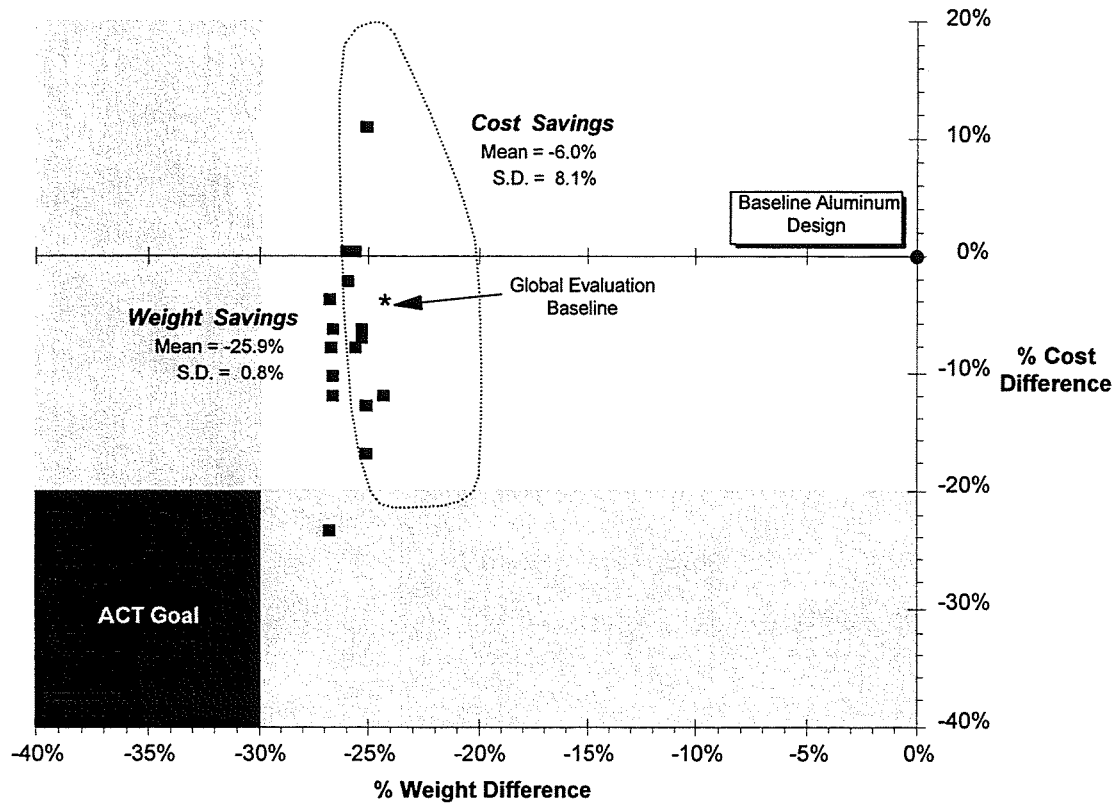


Figure 5-10. Composite Skin-Stringer Side Global DBT Cost & Weight Assessment

Team member inputs on the value of weight savings were combined with their cost and weight data to calculate total cost relative to the aluminum baseline. This combination of cost and weight data yields a single measure of team insights on the advantages of each composite concept versus the baseline aluminum. A smaller ratio indicates better value. Figure 5-12 shows the distribution of results for composite concepts. Averages and standard deviations for both concepts are also plotted to help visualize differences between the sandwich and skin-stringer designs. The average costs for the two concepts are within a standard deviation of each other. Again a slightly larger standard deviation suggests that the sandwich has greater risk associated with its additional potential.

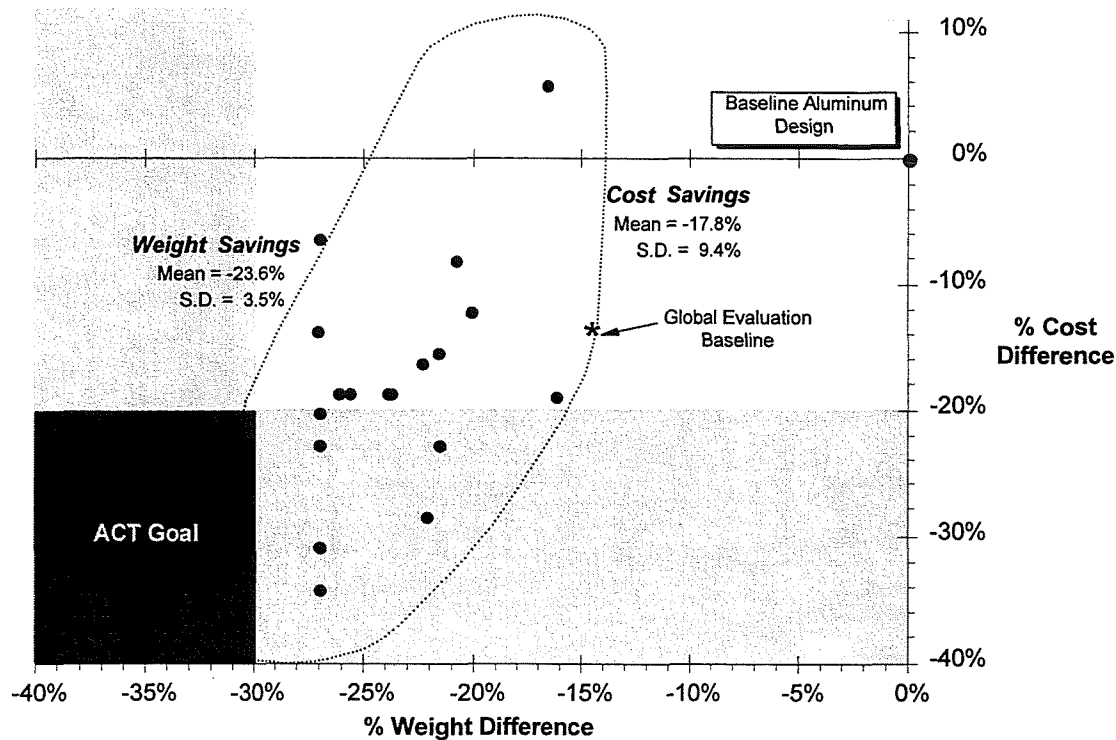


Figure 5-11. Composite Sandwich Side Global DBT Cost & Weight Assessment

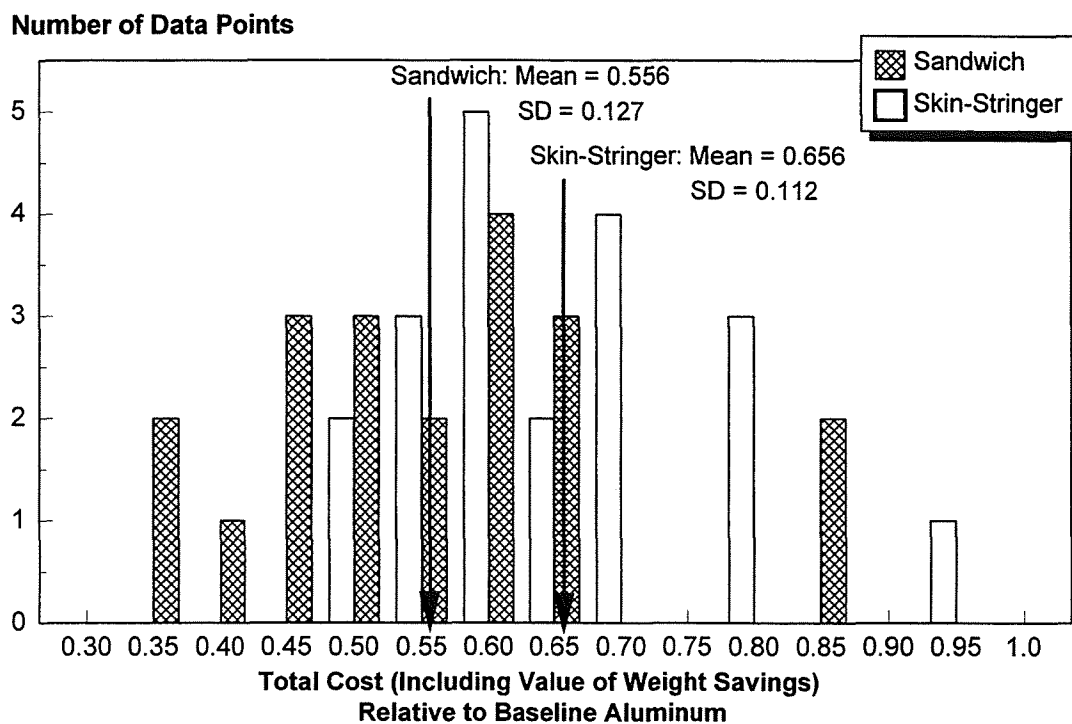


Figure 5-12. Distribution of DBT Data From Composite Skin-Stringer and Sandwich Evaluation Forms

5.5 Risk Reduction

The side panel global evaluation completes ATCAS Phase B trade studies to identify efficient composite fuselage structural configurations meeting ACT program goals. Design concepts, manufacturing processes, and related issues have been identified for fuselage crown, keel, and side panels in Section 46 of the study aircraft. As a result of these trade studies, sandwich configurations were identified to be most efficient for the keel (Ref. 2) and side panels. Through technical reviews of the side panel global evaluation trade study results with Boeing and NASA management, approval was obtained to continue pursuit of the cost savings potential of sandwich structure for transport fuselage under the ACT program. These reviews also confirmed the importance of weight savings which was deemed necessary to justify the additional risk of sandwich versus skin-stringer configurations.

Based on team member inputs and subsequent management reviews, a prioritized list of sandwich manufacturing and technical issues was developed to focus near-term ATCAS efforts for both the keel and side panel local optimization efforts. Figure 5-13 summarizes remaining Phase B efforts on technical risk reduction, local detailed design, and manufacturing scaleup for the selected side and keel sandwich concepts. The Phase B concepts and supporting databases will be re-evaluated at the start of ACT Phase C to facilitate design integration and scale-up for a full barrel manufacturing demonstration.

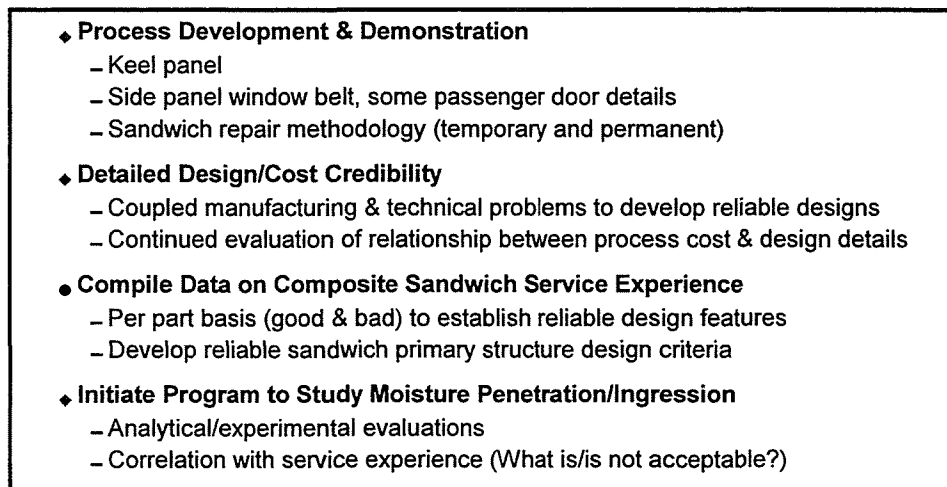


Figure 5-13. Sandwich Concept Development Efforts for the Remainder of Phase B

Each of the tasks listed in Figure 5-13 is directed towards demonstrating confidence in the use of sandwich transport fuselage structures. To further highlight the critical developments needed to achieve the desired level of confidence, the following technical issues were identified:

- Develop reliable & cost-effective sandwich keel & side panel design details
- Perform manufacturing scaleup & structural evaluation of sandwich keel (mid and forward keel panels) and side (window belt benchmark panels & door elements) hardware
- Develop repair methodologies suitable for airlines
- Perform liquid ingression analysis & test evaluation

- Begin development of cost-effective, light-weight sandwich panel close-outs
- Assess sandwich damage tolerance & bonded frame integrity

The remaining ATCAS Phase B side and keel panel studies will attempt to develop the database necessary to address each issue identified. The resolution approach and associated measures of success for each issue are defined in Figure 5-14.

Approach	Resolution
Keel & Side Panel Design <ul style="list-style-type: none"> • Compile data from past programs & in-service experiences for specific parts • Integrate results of process trials, building block tests, and COSTADE trades • Update keel, side panel design drawings based on local optimization results • Develop manufacturing cost relationships for selected design details 	<p>Documented cost & weight estimates for the optimized keel & side panel designs (incorporating results from all associated tasks) to quantify the sandwich cost/weight potential. Provide documented evidence that the selected design will be impervious to moisture/fluid ingress beyond immediate areas of damage.</p>
Keel Manufacturing & Structural Scaleup <ul style="list-style-type: none"> • Develop cure cycle for thick laminate (forward keel) & sandwich (aft keel) • Develop AFP methodology and cure tooling for mid & forward keel details • Demonstrate manufacturing processes by fabricating curved, 7' x 10' mid and forward keel panels • Evaluate manufacturing tolerances, associated assembly issues, & costs • Perform detailed analysis to assess keel load redistribution 	<p>Sandwich cure cycle development, advanced tow placement, and tooling for assembly, will be demonstrated by successfully processing several 7' x 10' mid and forward keel panels. Completed forward keel panels will be tested to demonstrate load redistribution.</p>
Side Manufacturing & Structural Scaleup <ul style="list-style-type: none"> • Develop AFP technology • Evaluate/develop window belt cure cycle & tooling • Demonstrate manufacturing methods at the scale of curved 7' x 10' window belt panels • Develop fabrication processes for selected door elements • Perform manufacturing tolerance measurements and analysis to assess mechanical assembly issues & costs • Perform detailed structural analysis to assess cutout performance 	<p>The development of a sandwich cure cycle, advanced tow placement process methodology, cure tooling concept and detailed structural analysis will be addressed by successful processing and testing of 7' x 10' demonstration panels.</p>
Sandwich Panel Repair <ul style="list-style-type: none"> • Develop overall approach to repair design & processing suitable for a range of damage scenarios (including airline insights & approval) • Explore suitable nondestructive evaluation procedures for completed repairs • Develop analysis tools and documentation • Initiate process trials at an airline and structural testing (NASA) 	<p>Demonstrate fuselage sandwich repair for a compression keel panel with thick laminate facesheets (30 ply), perform structural tests to evaluate supporting analyses, and obtain airline feedback on approach/design/process.</p>
Liquid Ingression Analysis & Test Evaluation <ul style="list-style-type: none"> • Literature review and in-service data collection • Define service environment • Identify & quantify ingress and internal propagation mechanisms • Determine effect of exposure on material properties • Evaluate response of cutout, splice design details by analysis & tests • Explore nondestructive evaluation procedures 	<p>Demonstrate the durability of keel and side panel materials and design details exposed to moisture and other fluids.</p>
Cost-Effective, Light-Weight Sandwich Edge Close-outs <ul style="list-style-type: none"> • Develop design concepts for sandwich panel splice close-outs • Explore suitable nondestructive evaluation procedures • Begin process development of candidate designs • Perform measurements and analysis to evaluate the dimensional stability and splicing costs of candidate designs • Initiate tests and analysis to evaluate joint load transfer and bypass stress conditions for candidate designs 	<p>Demonstrate processability and structural performance of selected sandwich panel close-out concepts with cost & weight savings potential for major structural splices.</p>
Sandwich Damage Tolerance & Bonded Frame Integrity <ul style="list-style-type: none"> • Perform CAI analysis and tests for sandwich panel designs • Perform large notch (tension & compression) analysis and tests for sandwich panel designs • Bonded frame pull-off analysis & tests (pressure pillowing and stability) for sandwich panel designs • Explore suitable nondestructive evaluation procedures • Combine structural database with viable NDE methods to define requirements for the keel & side design details studied 	<p>Demonstrate analysis & inspection methods suitable for scaling building block tests to predict damaged fuselage structural performance (load redistribution & residual strength as a function of damage extent) and bonded frame integrity for selected design details. Success in this task will build confidence in defining suitable design criteria, inspection procedures, allowables methods, and maintenance documentation.</p>

Figure 5-14. Approach and Resolution for Technical Issues to be Addressed Through Remainder of ATCAS Program (ACT Phase B)

6.0 CONCLUSIONS

Preliminary design of two fuselage side panel design concepts was performed. The designs differed significantly in material type, structural configuration, and manufacturing processes in order to distinguish a range of cost and weight variation for individual design features. The first design, designated Design B1, featured drape-formed J-section stringers cobonded to an automated fiber placed skin with mechanically fastened textile/RTM circumferential frames. The second design, designated Design D1, was a sandwich configuration featuring automated fiber placed facesheets, honeycomb core, and textile/RTM cobonded circumferential frames. Both designs were sized considering critical load cases, damage tolerance, attachment details, and fabrication processes. Engineering drawings defining the designs were generated, from which component weights were calculated. Detailed manufacturing and assembly preplans were also developed. Recurring and nonrecurring (excluding capital equipment) costs were estimated directly from the data contained in the preplans and engineering drawings. These estimates were generated in accordance with specific groundrules established for the ACT program. Based on the cost and weight estimates, the designs were refined to improve their efficiency. Finally, an assessment of further potential improvements which might be realized through additional development was performed.

Key design drivers were identified for both composite designs. Panel stability dictated the lower side panel configuration due to high shear loading at the forward end and combined shear-compression loading in the middle and aft regions. Damage tolerance was a concern over the majority of the middle and upper side acreage areas. In regions adjacent to door and window cutouts, ultimate strength was the critical factor. Panel edges were designed by splice requirements of bearing/bypass and minimum thickness for countersunk fasteners.

Weight results for the two composite global evaluation designs revealed that the B1 design offered the greatest savings, 23 percent, while the D1 design presented savings of 13 percent as compared with the baseline aluminum design. The skins were the major contributor, accounting for approximately 40 percent of the total weight of each design, as well as providing the majority of the weight savings over the baseline aluminum design. The circumferential frames and door structure also contributed to the lower weight of the composite designs as these elements were sized by ultimate strength whereas for the baseline aluminum design they were typically driven by fatigue considerations. All other elements were on par with the baseline aluminum design.

Cost estimates for the two composite designs revealed that nonrecurring costs comprise approximately 25 percent of the total with the remaining fraction fairly evenly distributed between recurring material, fabrication labor, and assembly labor. In terms of design elements, skin panel assembly and final panel assembly/installation comprised approximately 43 percent of the total cost of the composite designs. This is due to the large size of the ATCAS side panel and the numerous complex subassemblies (i.e., window frames, door structure, and fuselage frames). Virtually all of the cost savings achieved by the composite designs was attributable to skin fabrication with the composite designs benefiting from the highly automated fiber placement process. Comparative results indicate the D1 design offers the greatest cost savings (13 percent) over the baseline aluminum while the B1 design showed a 4 percent savings.

Global optimization of the composite design concepts was performed by trading specific design details to gain performance advantages or reduce costs. Two examples of this are the use of pultruded stringers in lieu of drape-forming to reduce the B1 fabrication cost and replacing the 8.0-3/16 HRP core with DuPont's 4.5-1/8 Korex on the D1 design to reduce weight. As a result of this exercise, cost savings (relative to the baseline aluminum) for the B1 design increased to 7 percent while the D1 design's cost savings increased to 16 percent. Weight savings for the B1 design improved slightly to 25 percent while the globally optimized D1 design achieved 22 percent weight savings.

As the final step in the side panel global evaluation, the potential to further improve the composite designs during local optimization was assessed. Validation of these design modifications was acknowledged to require significant development but these trades were important in answering "what if?" questions regarding the influence of selected design details. For the B1 design, hat instead of J-section stringers and lower part deformation leading to reduced shimming requirements on assembly were considered. The D1 design was shown to benefit from less conservative damage tolerance criteria and increased circumferential frame spacing as well as reduced shimming requirements. The results of this final assessment placed the potential cost savings of the B1 design at 23 percent and the D1 design at 40 percent relative to the baseline aluminum. Similarly, potential weight savings were projected at 23 percent for the B1 design and 29 percent for the D1 design.

Based on the cost and weight results and assessment of the risks associated with each design, the D1 design was selected for continued development during side panel local optimization. The risks considered in making this selection included those associated with manufacturing process development, performance characteristics, and the ability to demonstrate the chosen concept within the confines of available program resources. The key to selection of the D1 design was the greater potential it offered to meet the ACT program goals. An additional outcome of the side panel global evaluation process was identification of critical design issues for each design. These issues will focus the D1 local optimization efforts such that the major design details will be quantified in terms of structural performance and manufacturing cost.

During local optimization, the next and final stage of side panel development, the sandwich design will be further refined. This will include results of manufacturing scaleup activities (directed primarily towards the window belt area) and quantification of the cost and weight potentials identified during the global evaluation phase. More extensive design tailoring will be pursued for the major side panel components; the NASA/Boeing Cost Optimization Software for Transport Aircraft Design Evaluation (COSTADE) will be utilized to support optimization of design variables such as laminate and core thicknesses, ply orientations, and stacking sequences. Significant resources will also be directed towards development and demonstration of side panel fabrication processes, with particular emphasis on the window belt region. Skin and core materials selected during global evaluation will be characterized to determine structural performance and processing parameters. In addition, the environmental durability and structural integrity of key design details such as sandwich panel closeouts and bonded elements will be evaluated. Study of all these issues will provide the confidence and database necessary to carry the side panel sandwich design into full scale development, to be performed during Phase C of the ACT program.

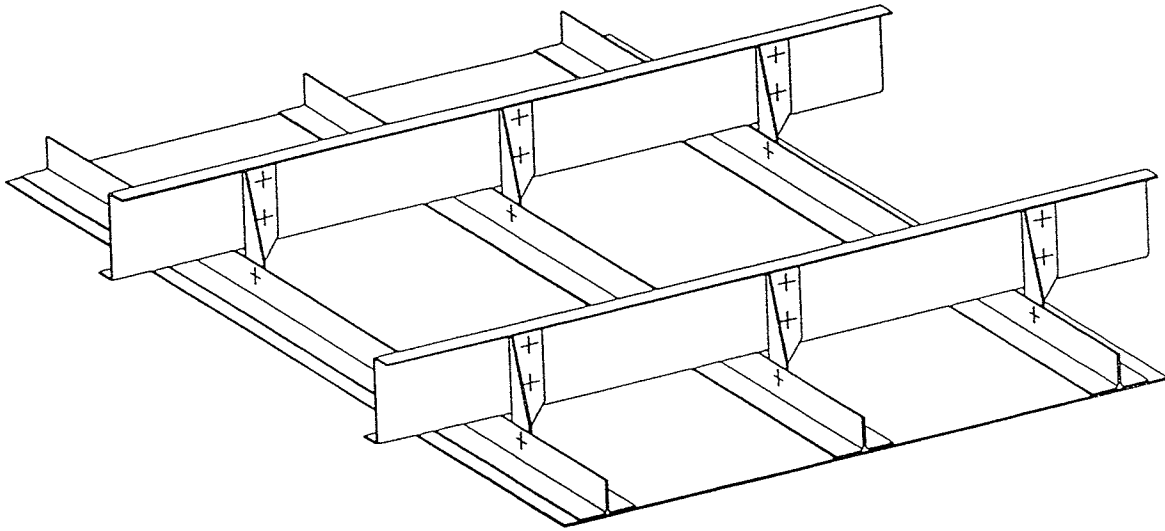
7.0 REFERENCES

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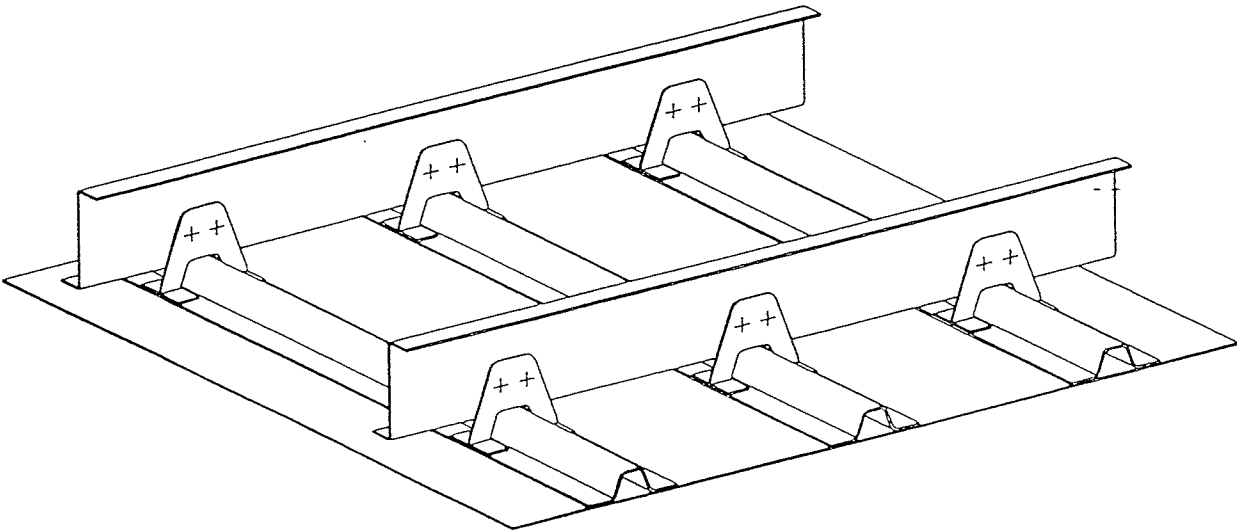
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APPENDIX A

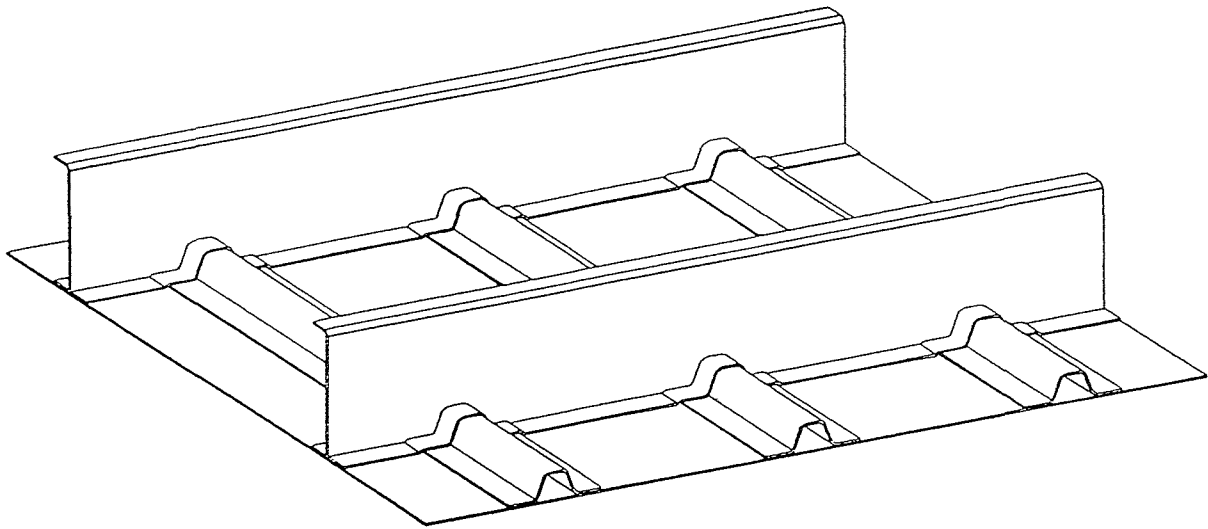
DESIGN FAMILIES



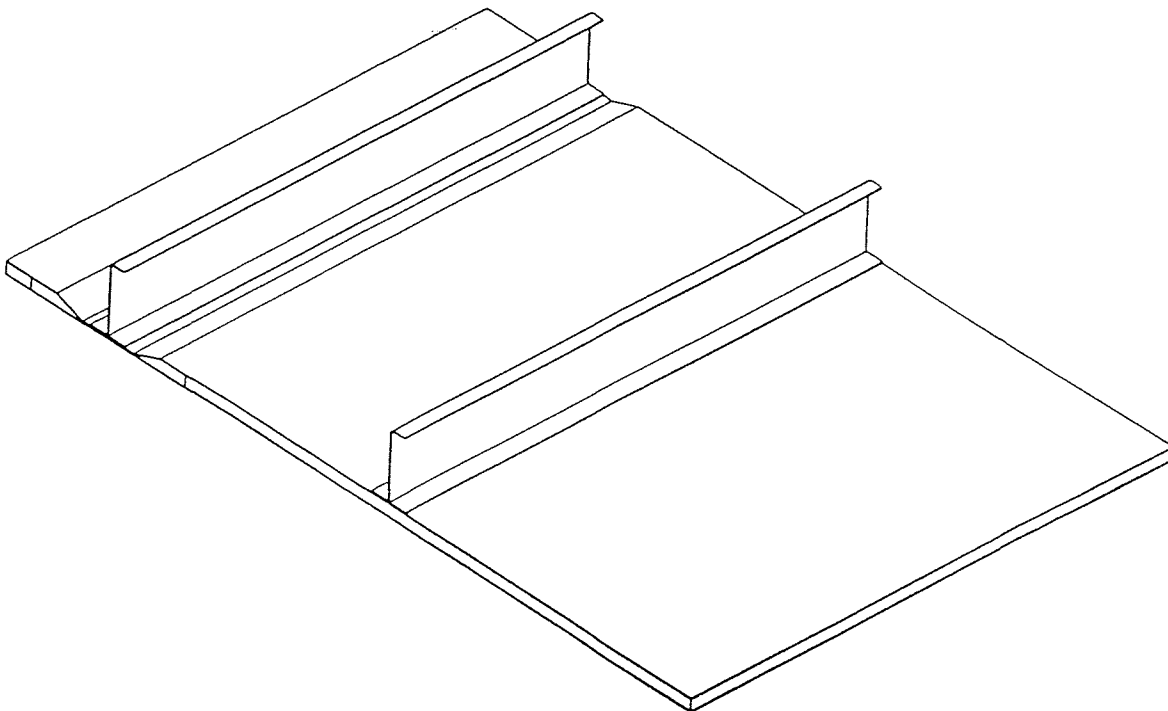
Family A: Skin-Stringer-Frame (Mechanically Fastened Stringers and Frames)



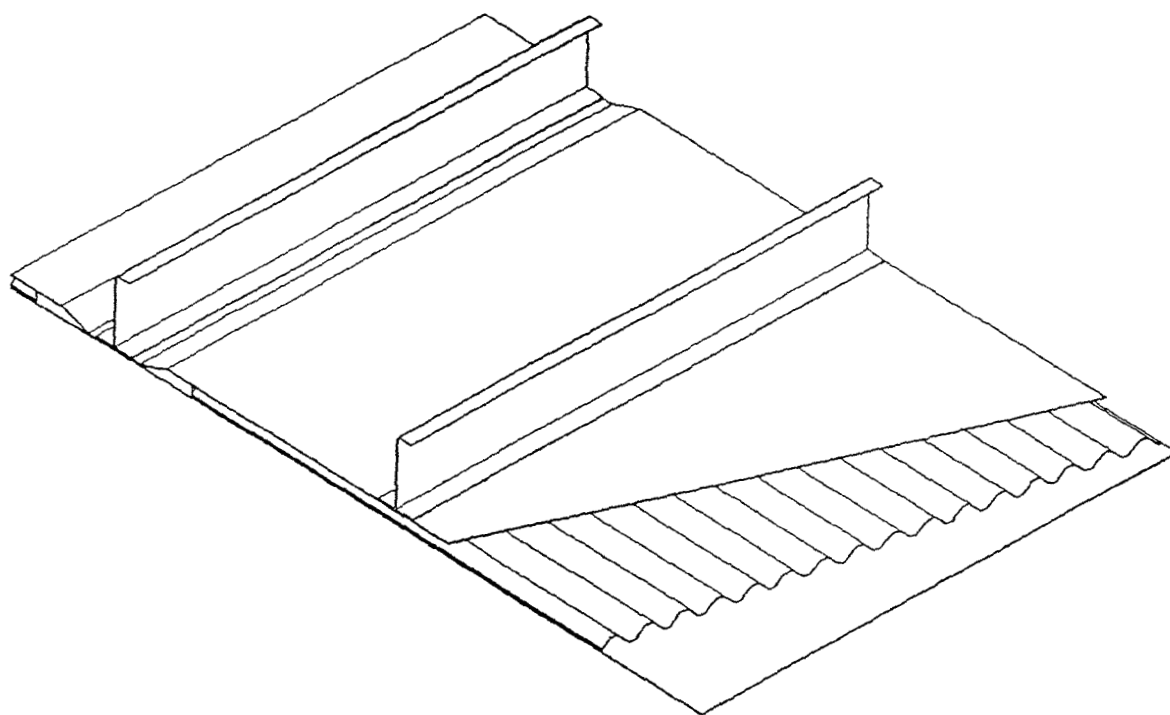
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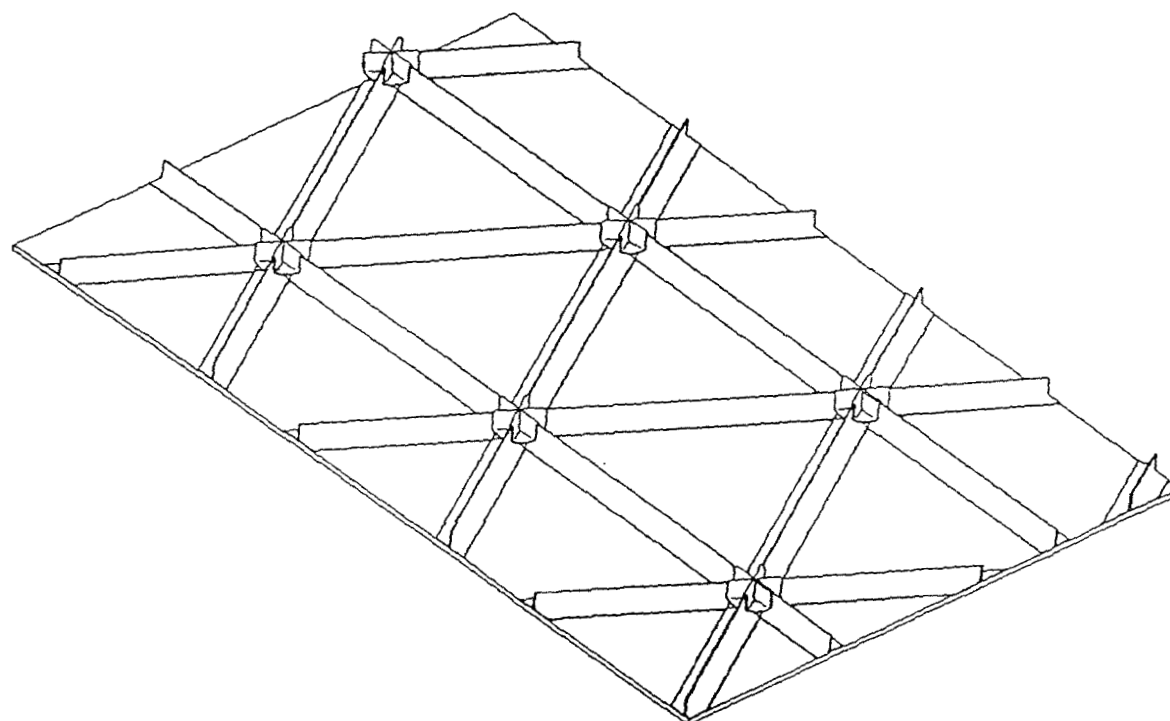
Family C: Skin-Stringer-Frame (Bonded Stringers and Frames)



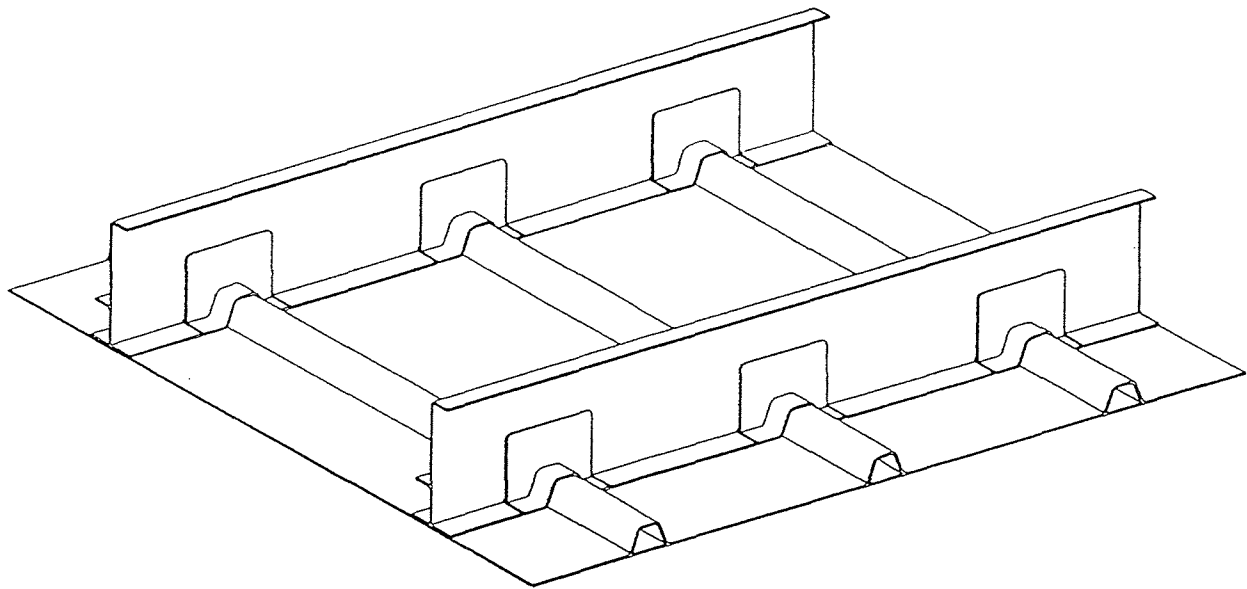
Family D: Sandwich



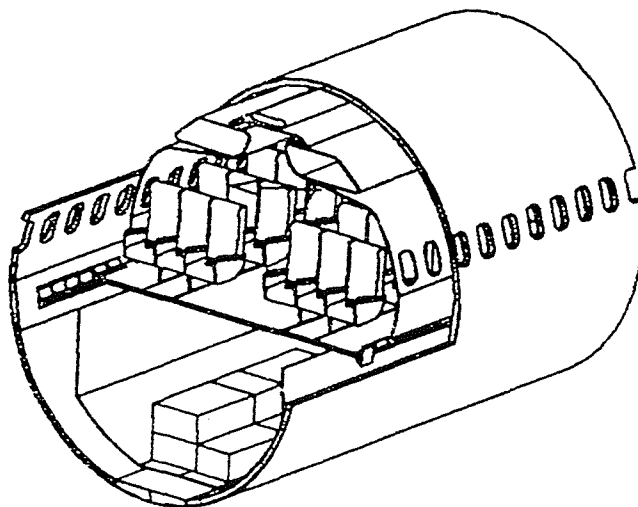
Family E: Corrugated



Family F: Geodesic



Family G: Integrally Stiffened Skins

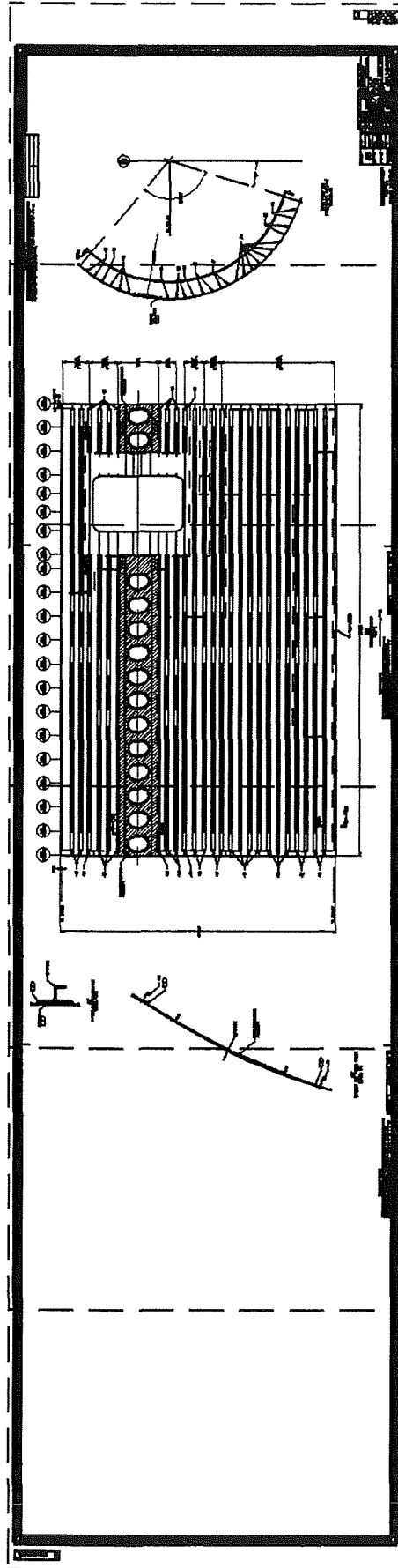


Family H: Continuous 360°

APPENDIX B

FAMILY B DESIGN DRAWINGS

B-1



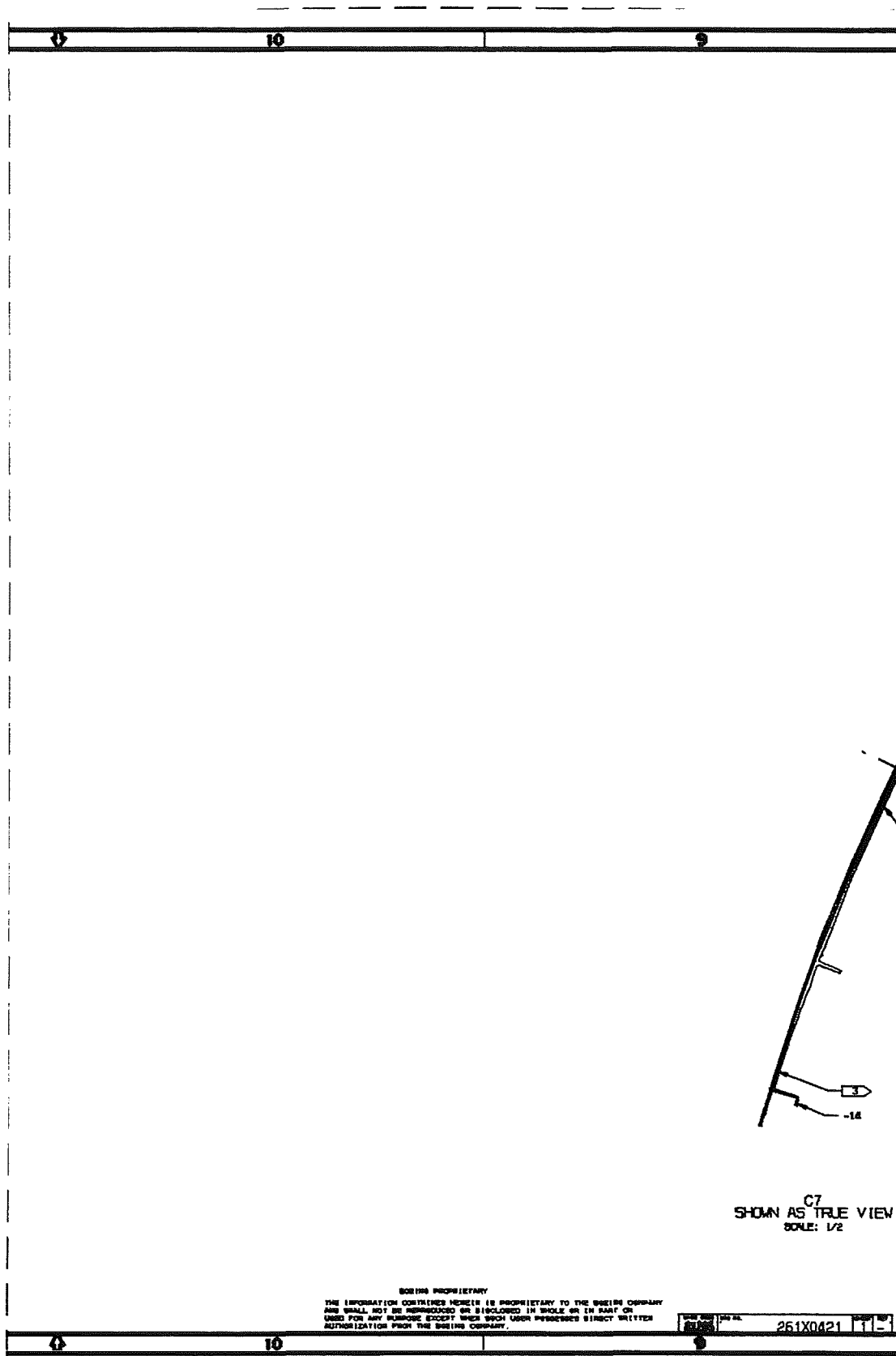
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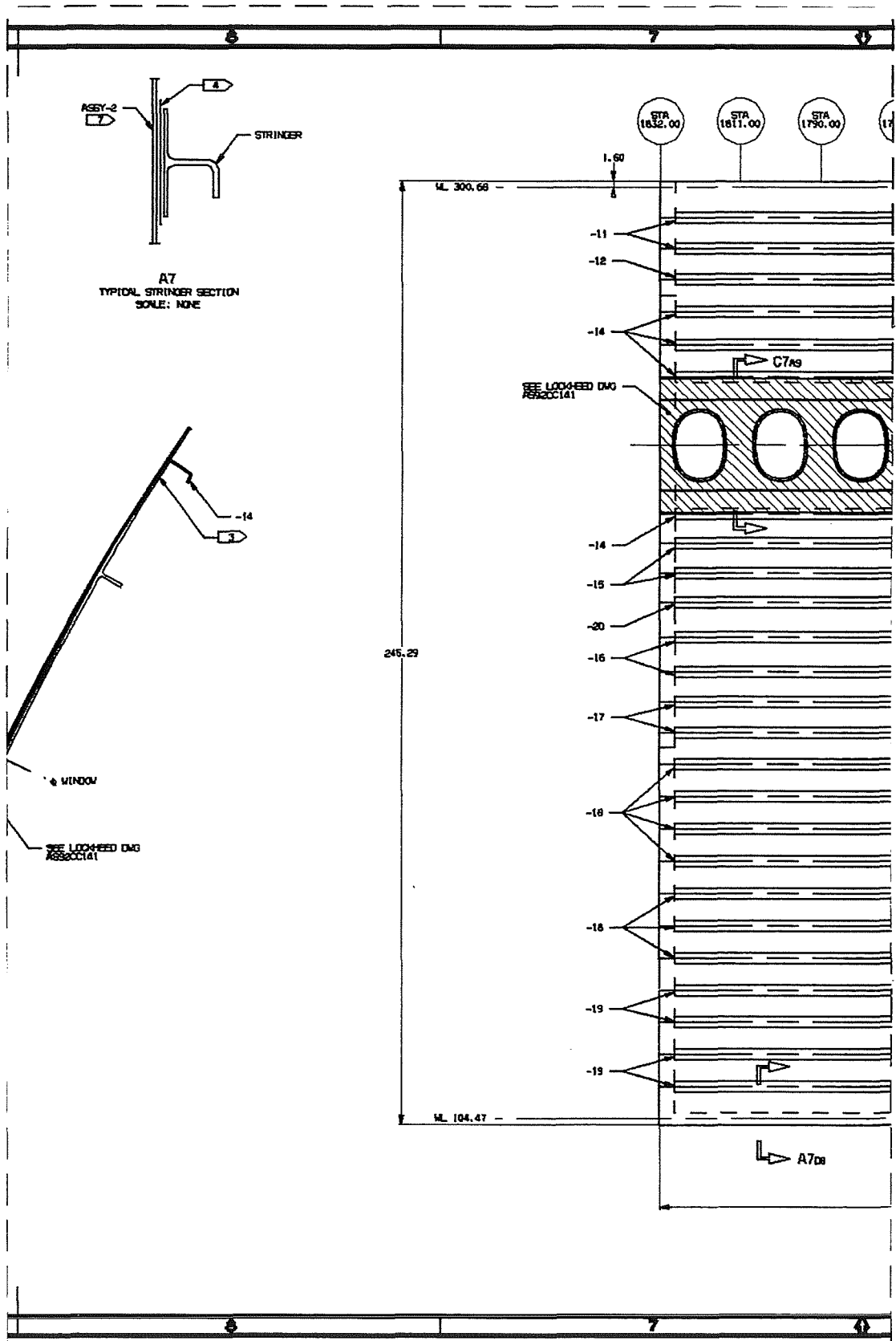
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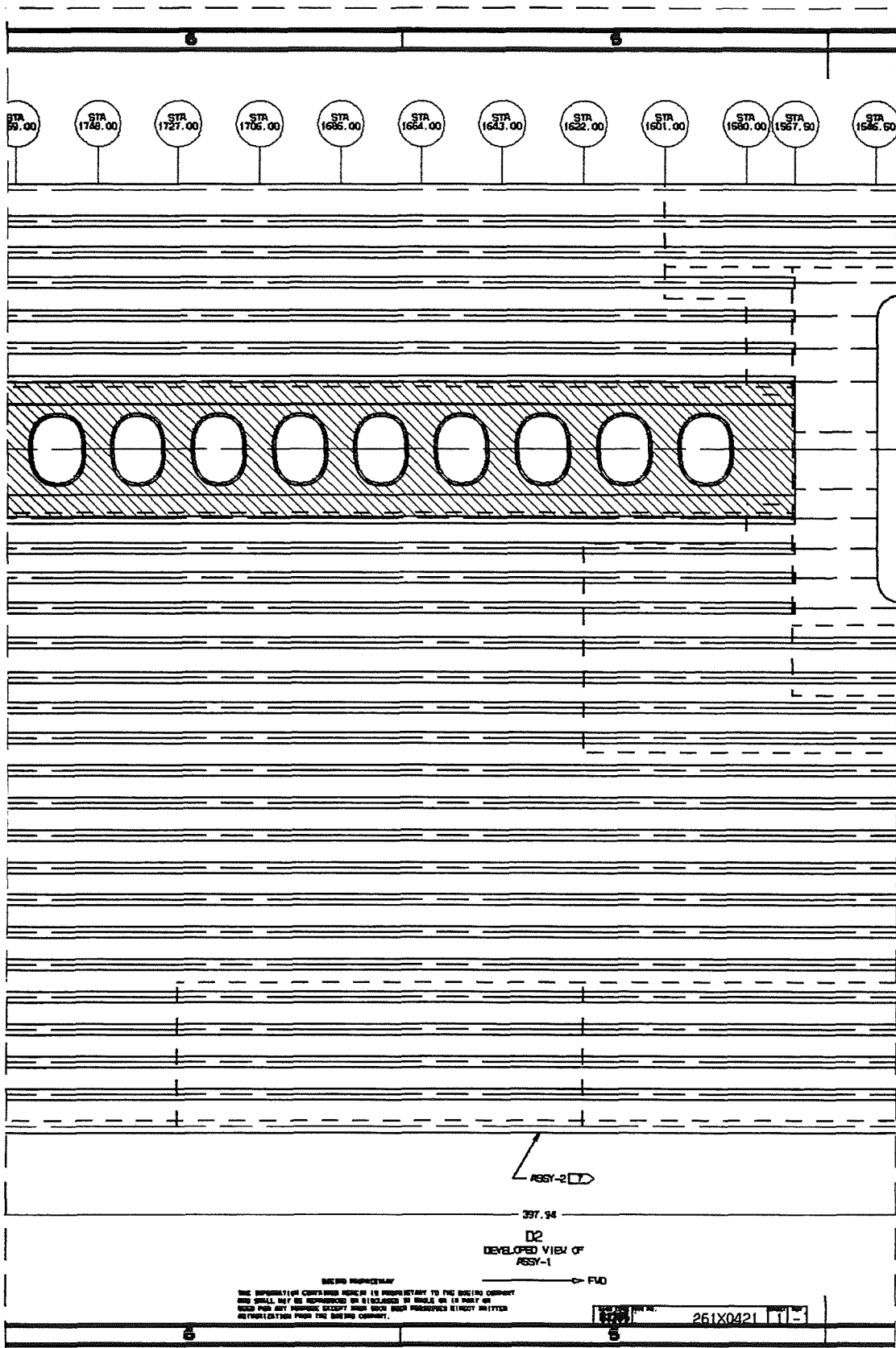
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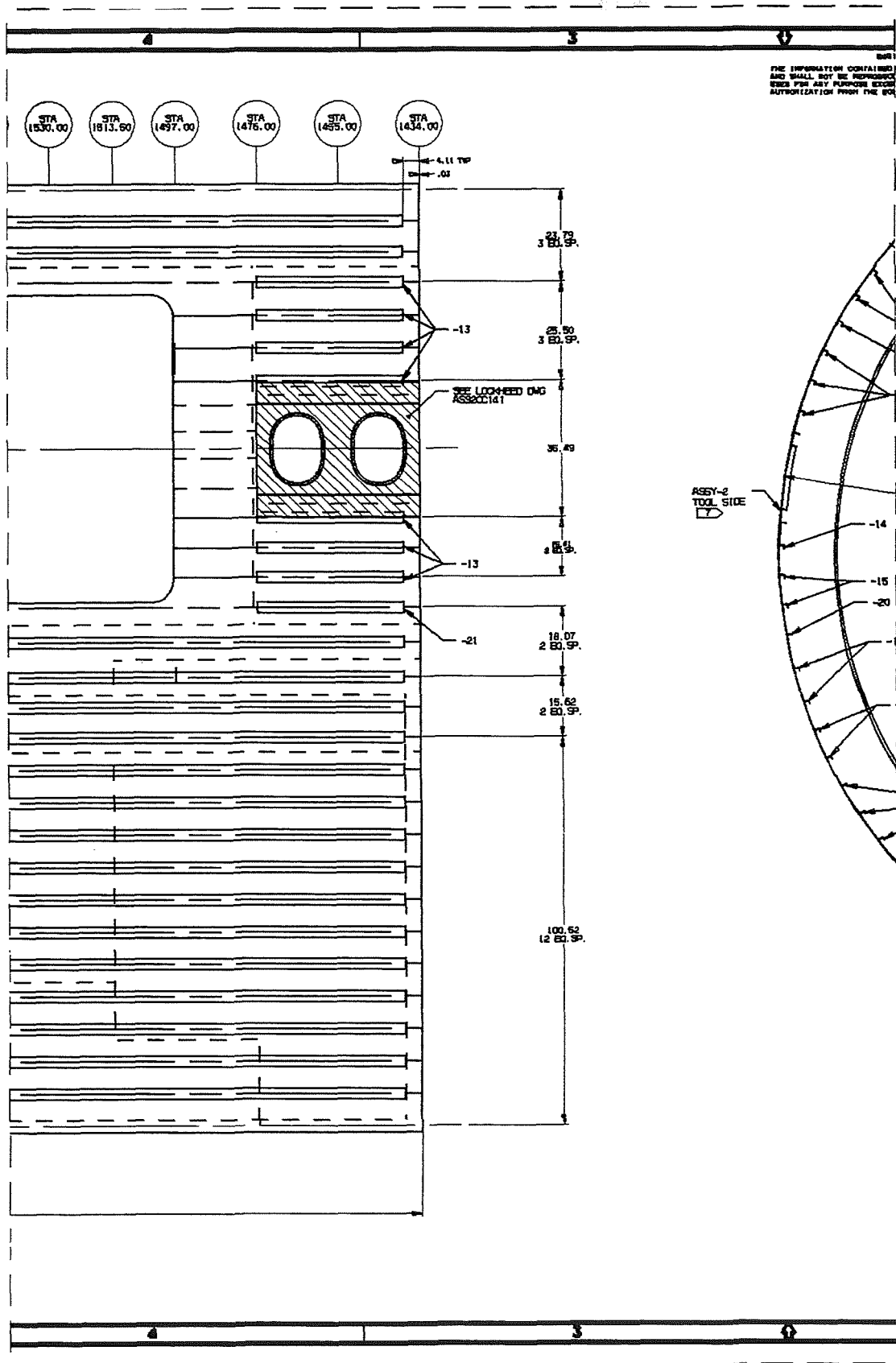
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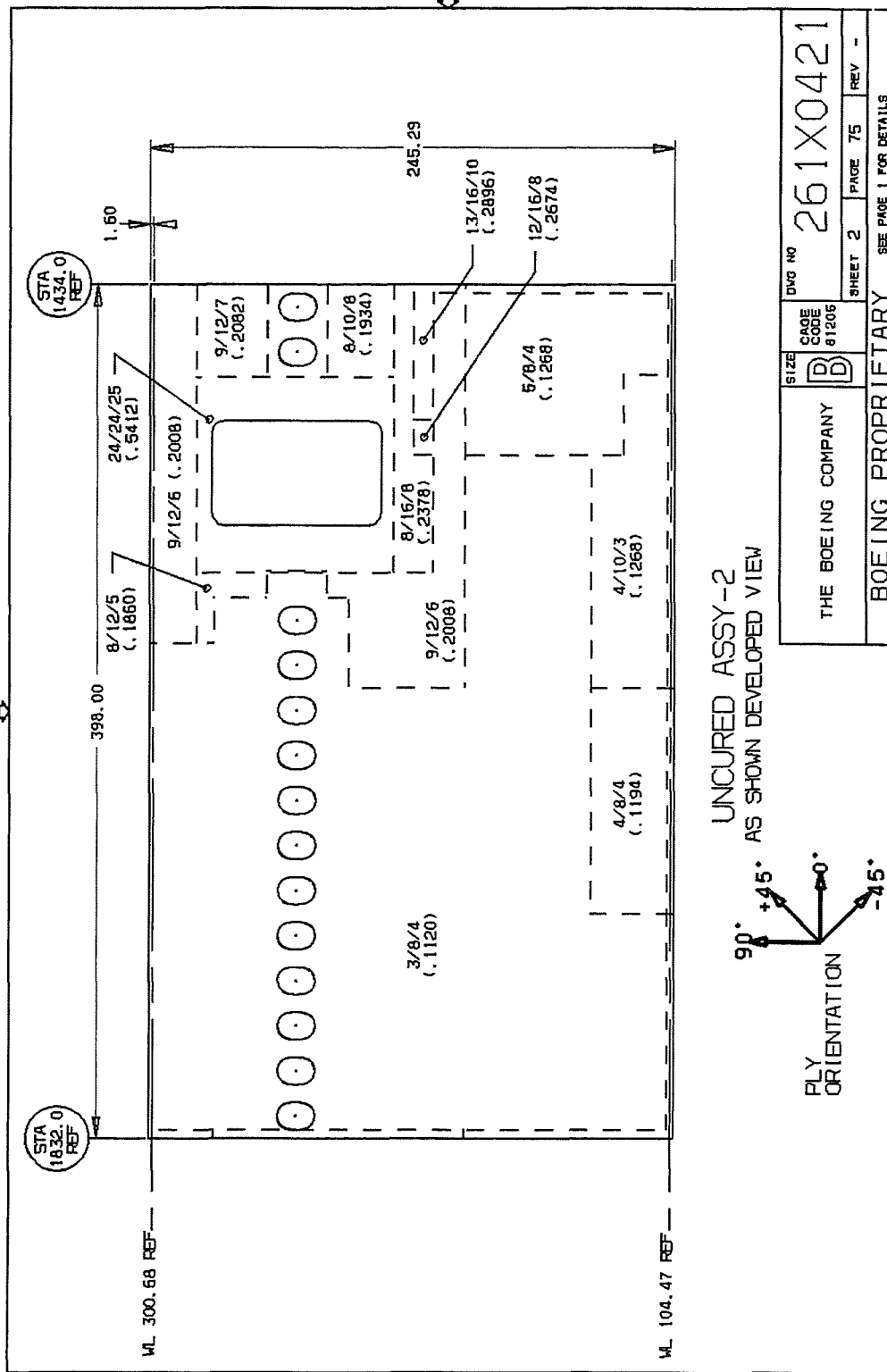
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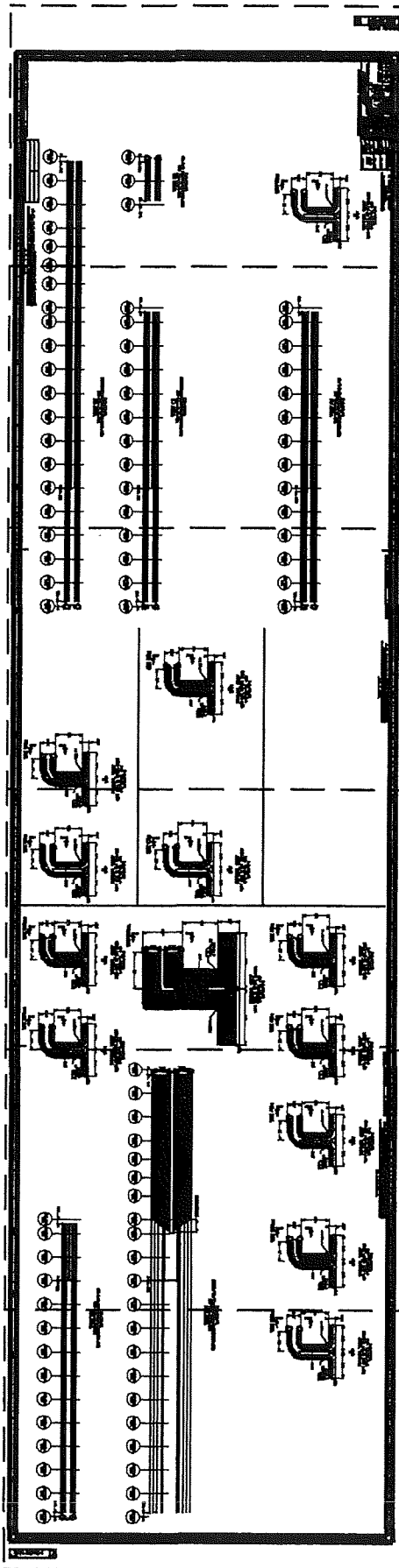






DI 5007 3760 CAT1A REV 8/92

THE BOEING COMPANY	SIZE B D	CASE CODE 81206	DWG NO 261X0421	SEE PAGE 1 FOR DETAILS		
				SHEET 2	PAGE 75	REV -
BOEING PROPRIETARY						



B-9

B-10

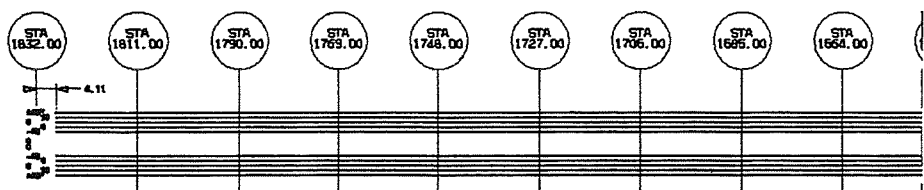
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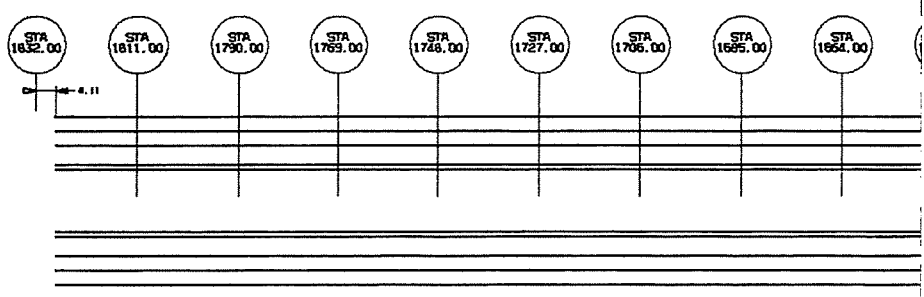
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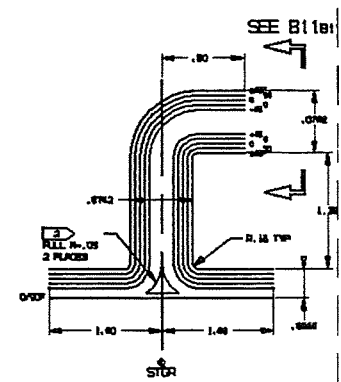
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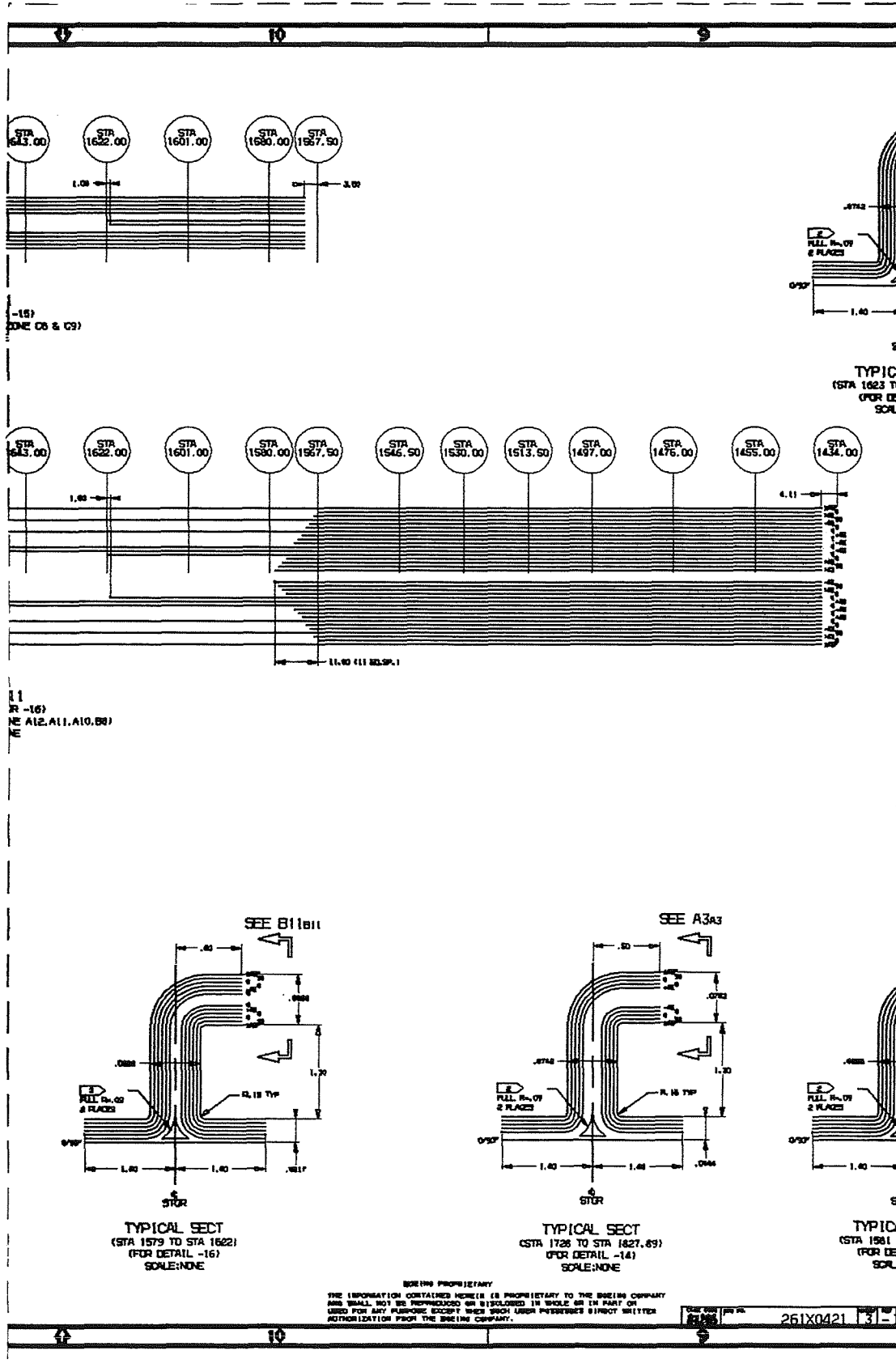


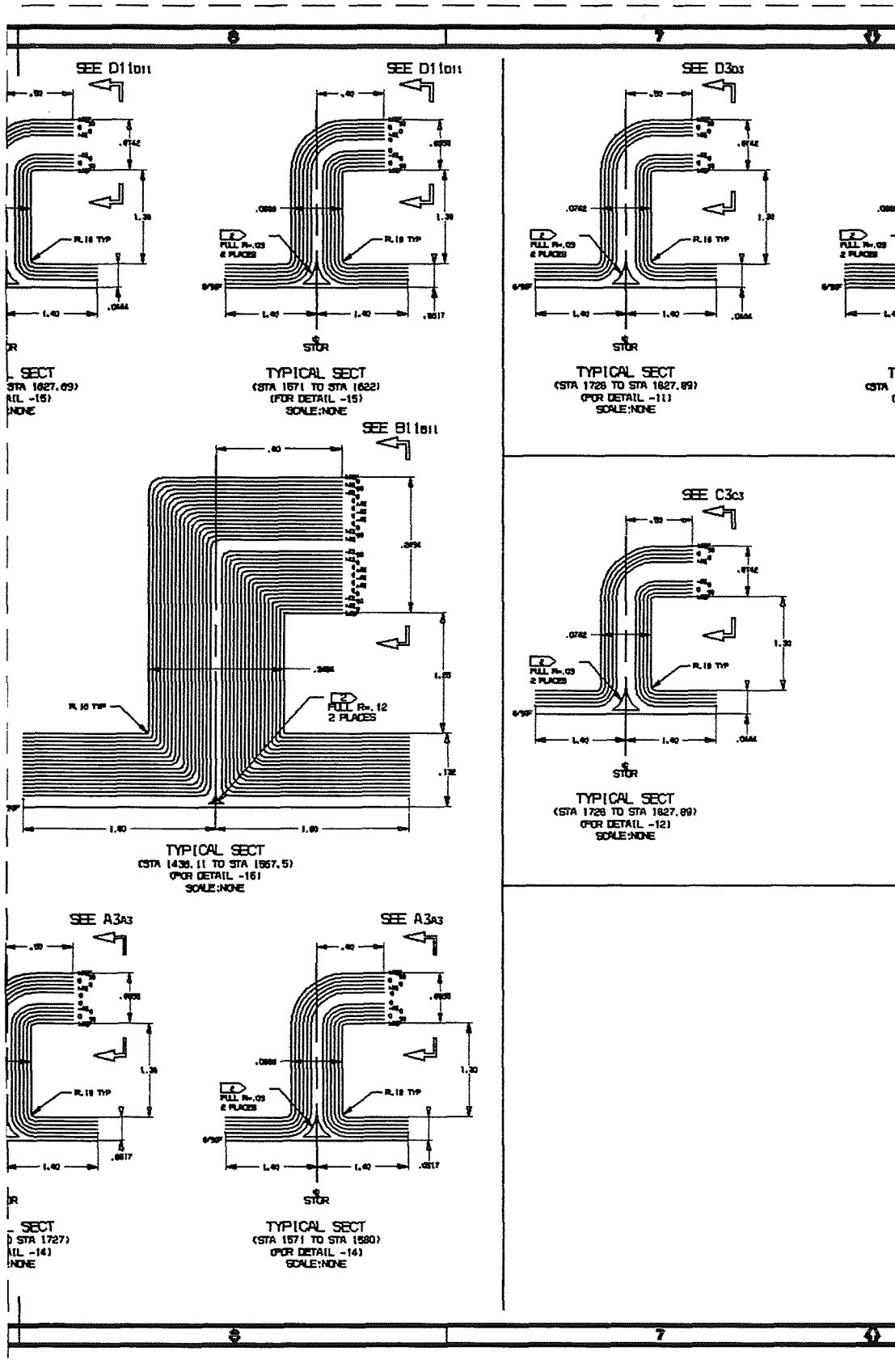
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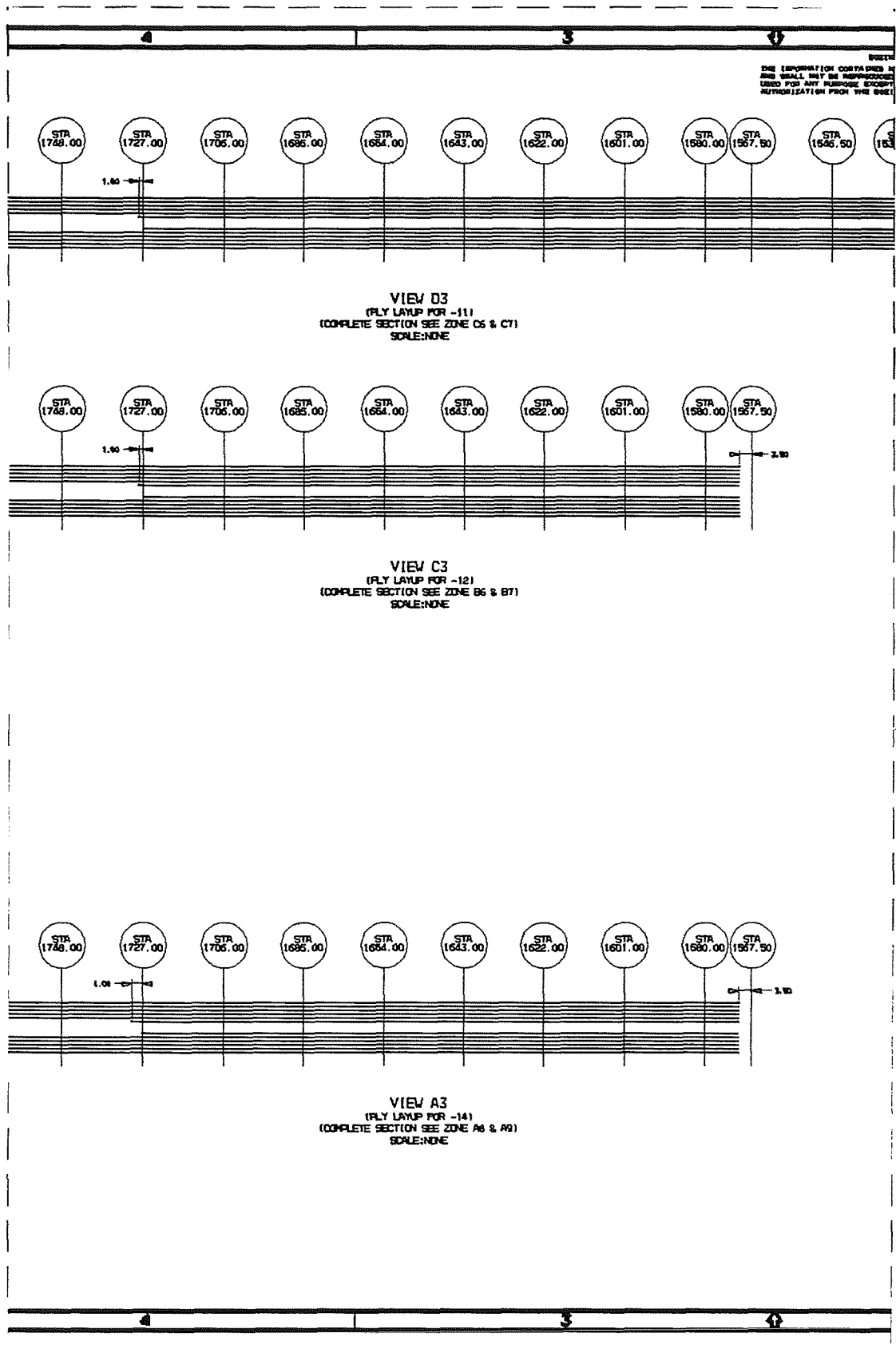


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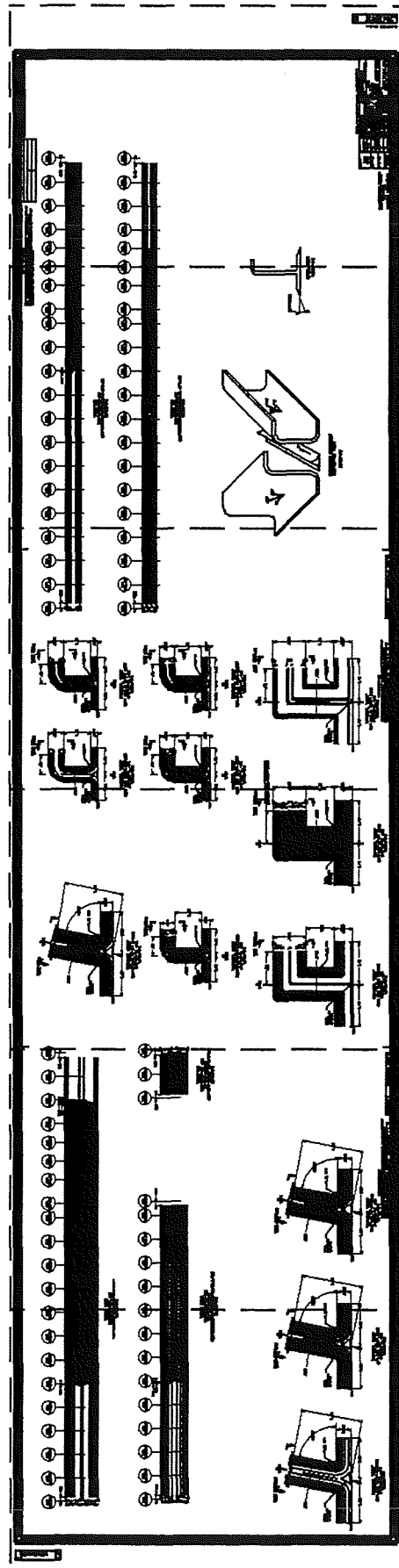








B-15



B-16

B-17

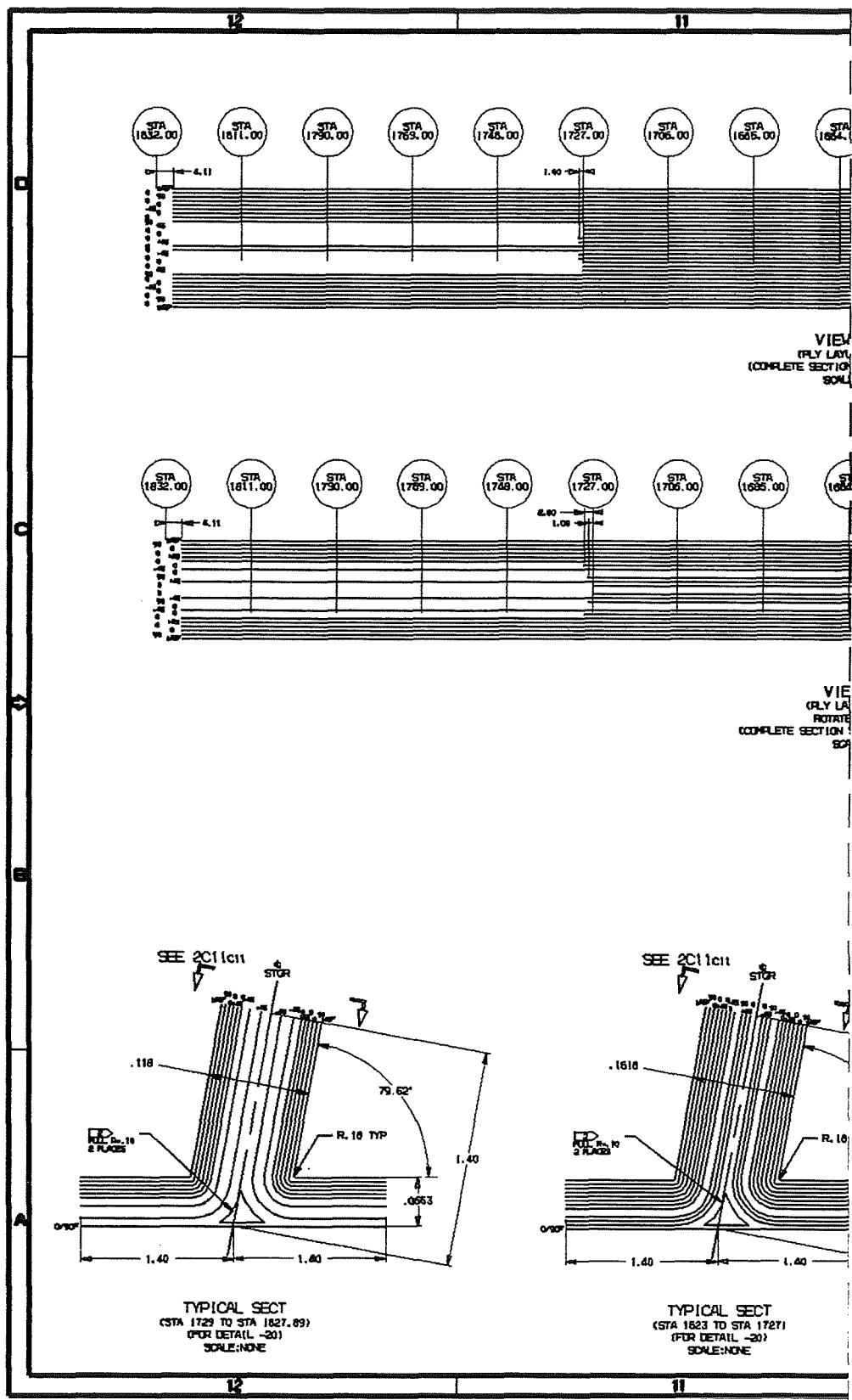
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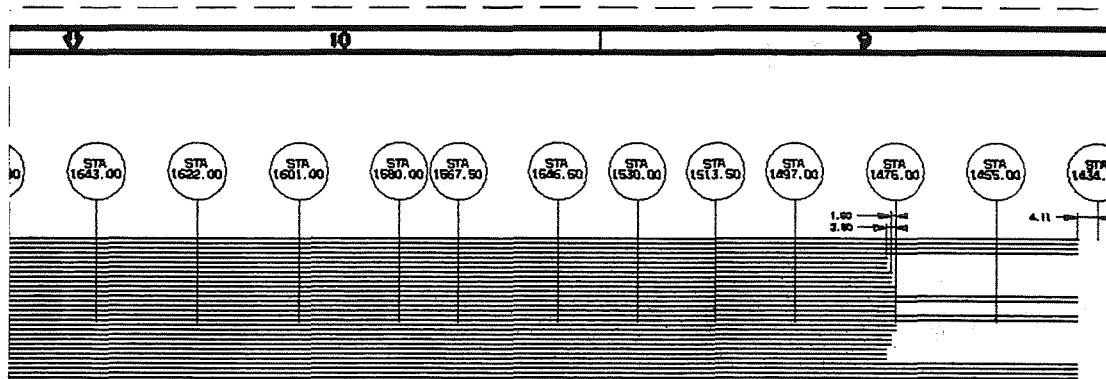
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B-20

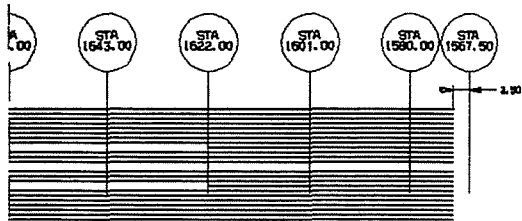
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4

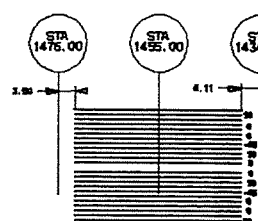




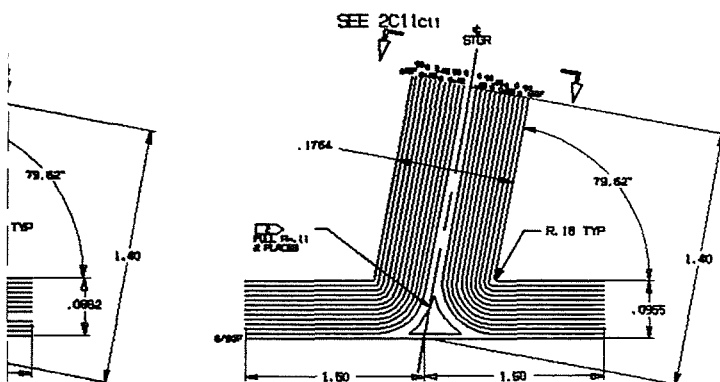
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P FOR -19)
SEE ZONE C5 & C7)
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W 2C11
UP FOR -20)
D 79.62' CV
SEE ZONE A10, A11, & A12)
SCALE: NONE



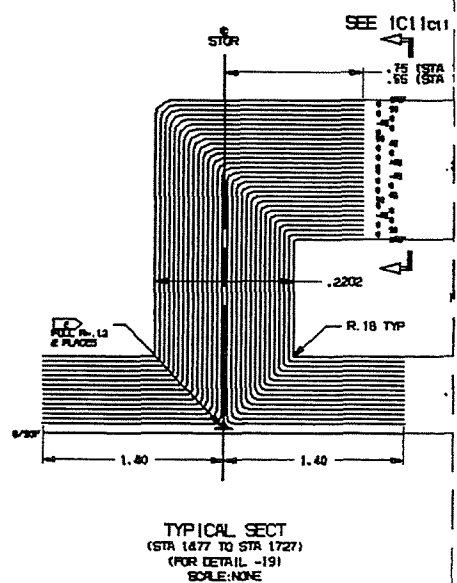
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ROTATED 79.62' CV
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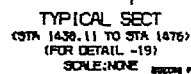
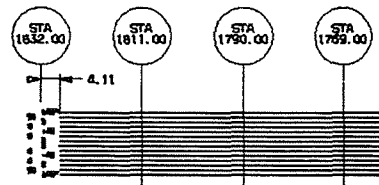
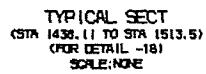
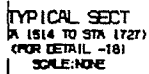
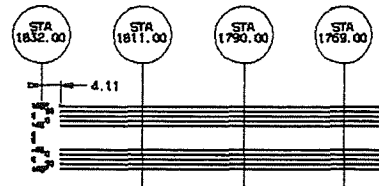
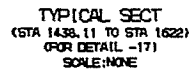
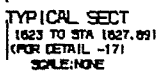


TYPICAL SECT
(STA 1571 TO STA 1622)
OFOR DETAIL -20)
SCALE: NONE

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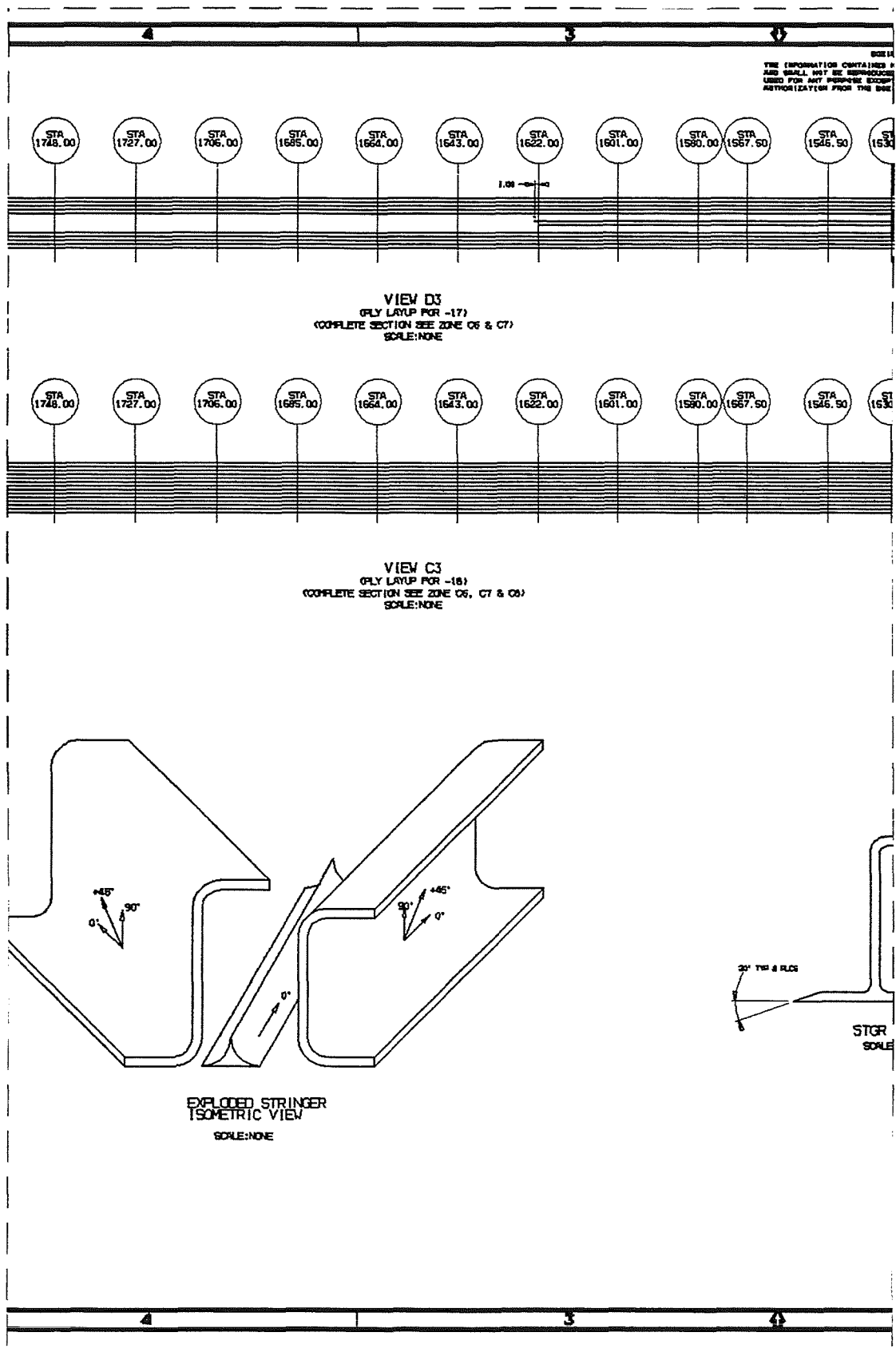
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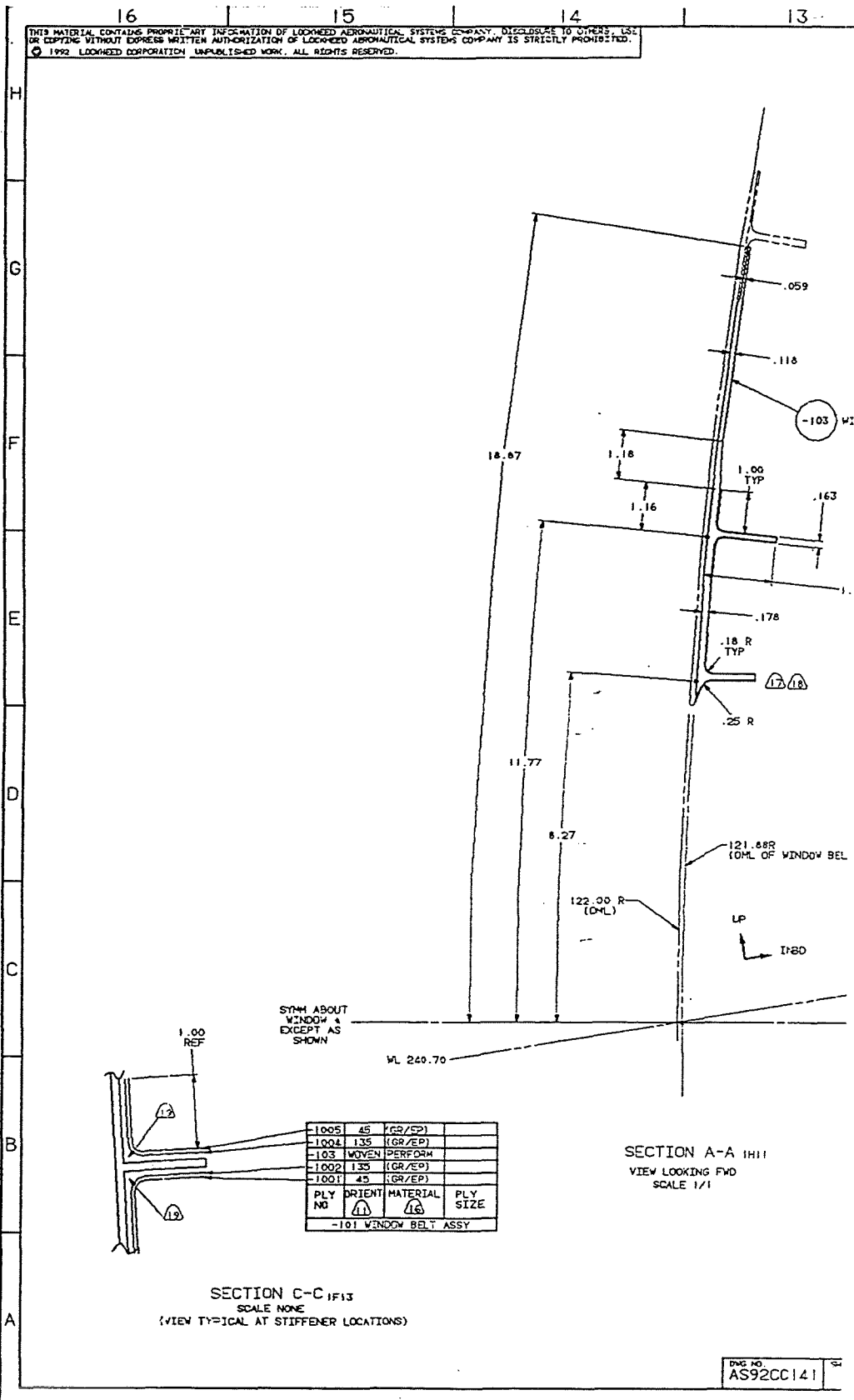


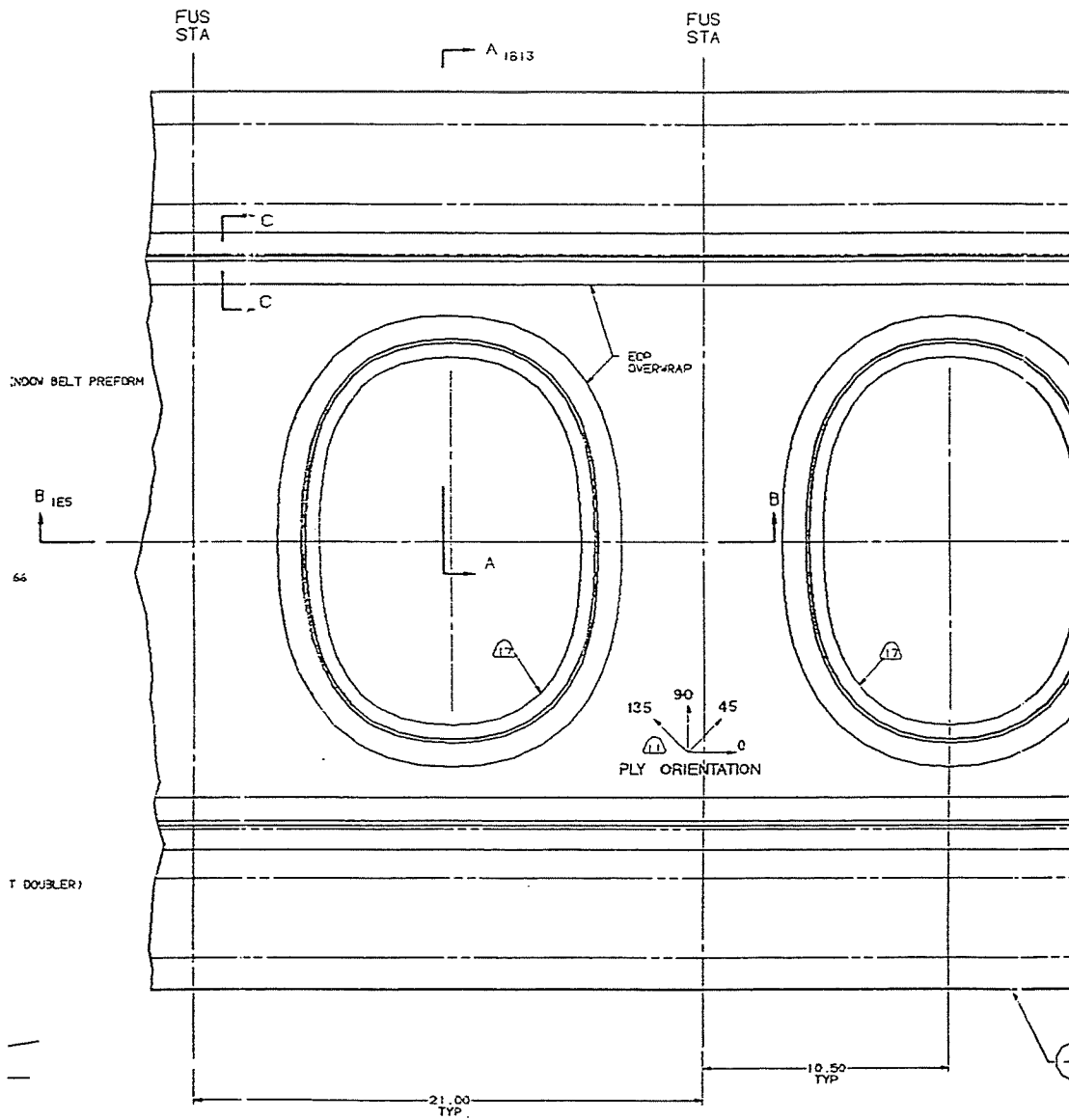


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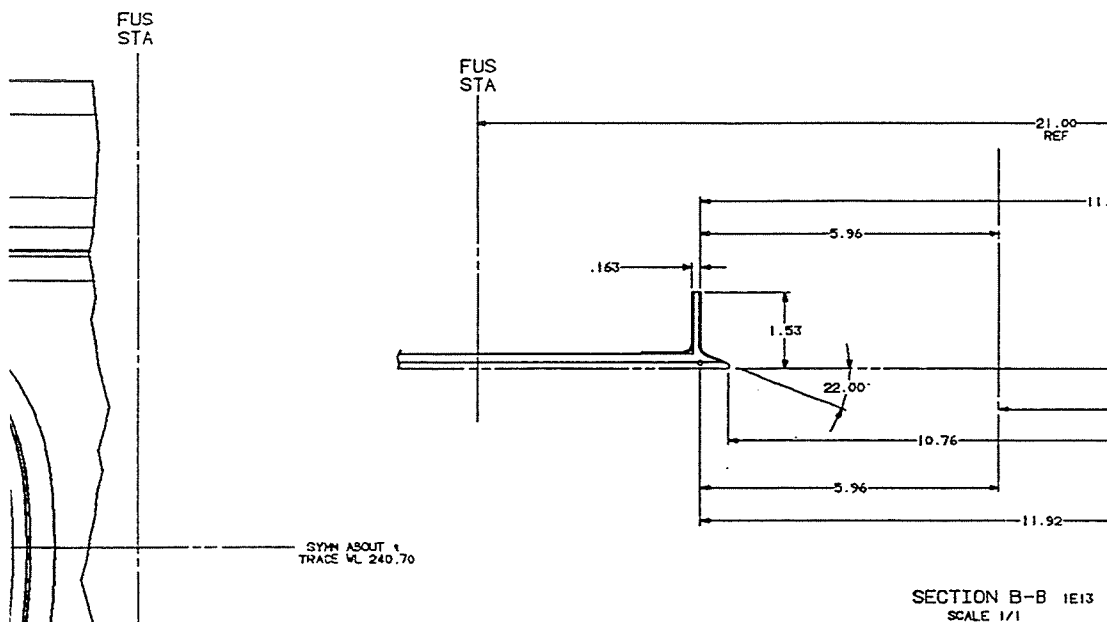




VIEW LOOKING OUTBD, LHS

-103 PANEL PREFORM ARCHITECTURE TABLE

PANEL AREA	0° FIBER ANGLE			135° FIBER ANGLE			90° FIBER ANGLE			THROUGH THE THICKNESS FIBER		
	MATL TYPE	TOW SIZE	FIBER %	MATL TYPE	TOW SIZE	FIBER %	MATL TYPE	TOW SIZE	FIBER %	MATL TYPE	TOW SIZE	FIBER %
SKIN	13	6K	25	13	6K	50	13	6K	25	13	2 PLY	1.0
STIFF	13	6K	82	---	---	0	13	6K	18	13	2 PLY	2.0

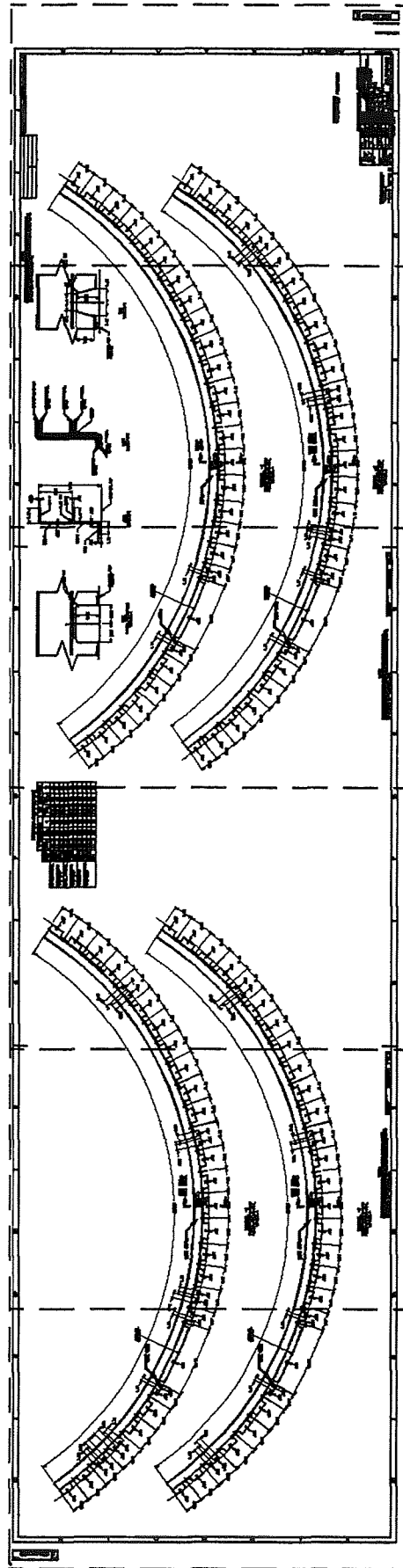


- 13 STUFFER REQUIRED. MATERIAL 13 OR EQUIVALENT
- 18 0° FIBERS ARE IN CIRCUMFERENTIAL DIRECTION FOR THE WINDOW FRAME
- 17 FOR WINDOW CUTOUT SEE LOFT DRAWING
- 16 MATERIAL: AS-4 OR G-30-500/AM00036 GRAPHITE/POWDERED EPOXY TOW/REG FABRIC, 6K TOW SIZE, 11 ENDS/INCH. (CURED THICKNESS .015 IN/PLY)
- 15 STIFFENER PARALLEL TO WINDOW
- 14 MATERIAL: TORAY T900 DRY GRAPHITE FIBER TOW.
- 13 MATERIAL: AS-4 OR G-30-500/AM00036 GRAPHITE/POWDER EPOXY TOW/REG.
- 12 FABRICATE PER ASM-CP-2 SPECIFICATION.
- 11 PLY ORIENTATION SYMBOL
- 10 LAST VIEW OR SECTION LETTER C.
- 9 LAST PLY NUMBER IS -1004.
- 8 QA INSPECT PER 6P1PCT01.
- 7 PLY SIZE AND EXTRA SCRAP DETERMINED BY MANUFACTURING.
- 6 PLY ORIENTATION SYMBOL SHOWN ON THIS DWG. SHOULD BE USED. (DISREGARD THOSE SYMBOLS SHOWN IN DRAFTING PRACTICE MANUAL, ENGINEERING DESIGN HANDBOOK, MANUFACTURING PROCESS SPECIFICATION, ETC.)
- 5 IMPRESSION STAMPING NOT PERMITTED
- 4 REGARDING STANDARD PARTS: REFER TO ENGINEERING STANDARDS MANUAL VOLUME 2 FOR APPROVED INTERCHANGEABLE EQUIVALENTS, SUPERSEDING PARTS, SOURCES, AND PROCUREMENT/QUALITY ASSURANCE REQUIREMENTS.
- 3 ABBREVIATIONS PER DRAWING 152893 AND MIL-STD-12.
- 2 ODD DASH NUMBER SHOWN NEXT HIGHER CONSECUTIVE EVEN DASH NUMBER-OPPOSITE.
- 1 THIS DRAWING WAS PREPARED BY COMPUTER PLOT. SEE RM FOR SIGNATURES.

NOTE:

CADAM DWG
CAD MAINTAINED CHANGES
SHALL BE INCORPORATED BY
THE DESIGN ACTIVITY.

FRAM



B-27

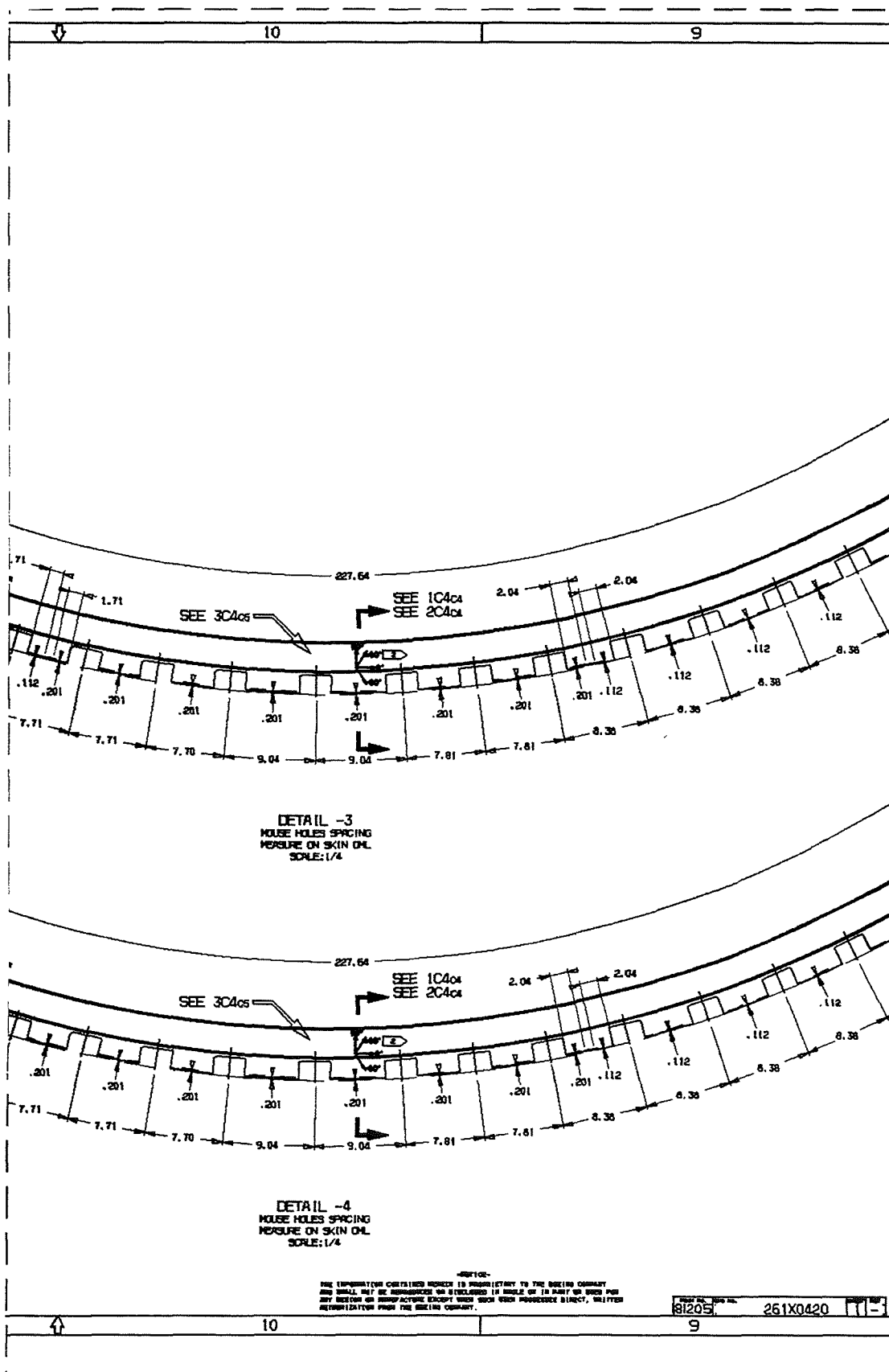
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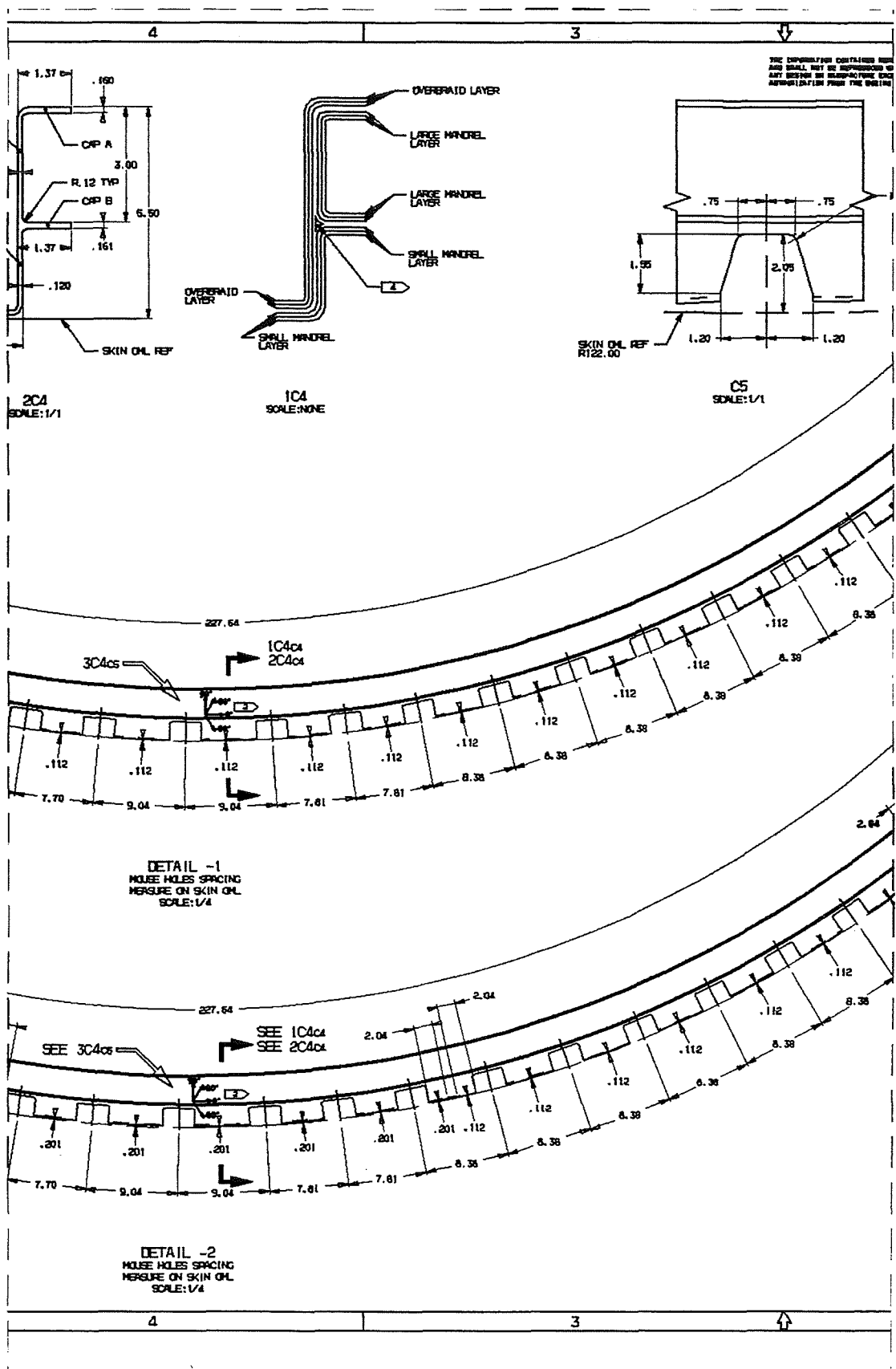
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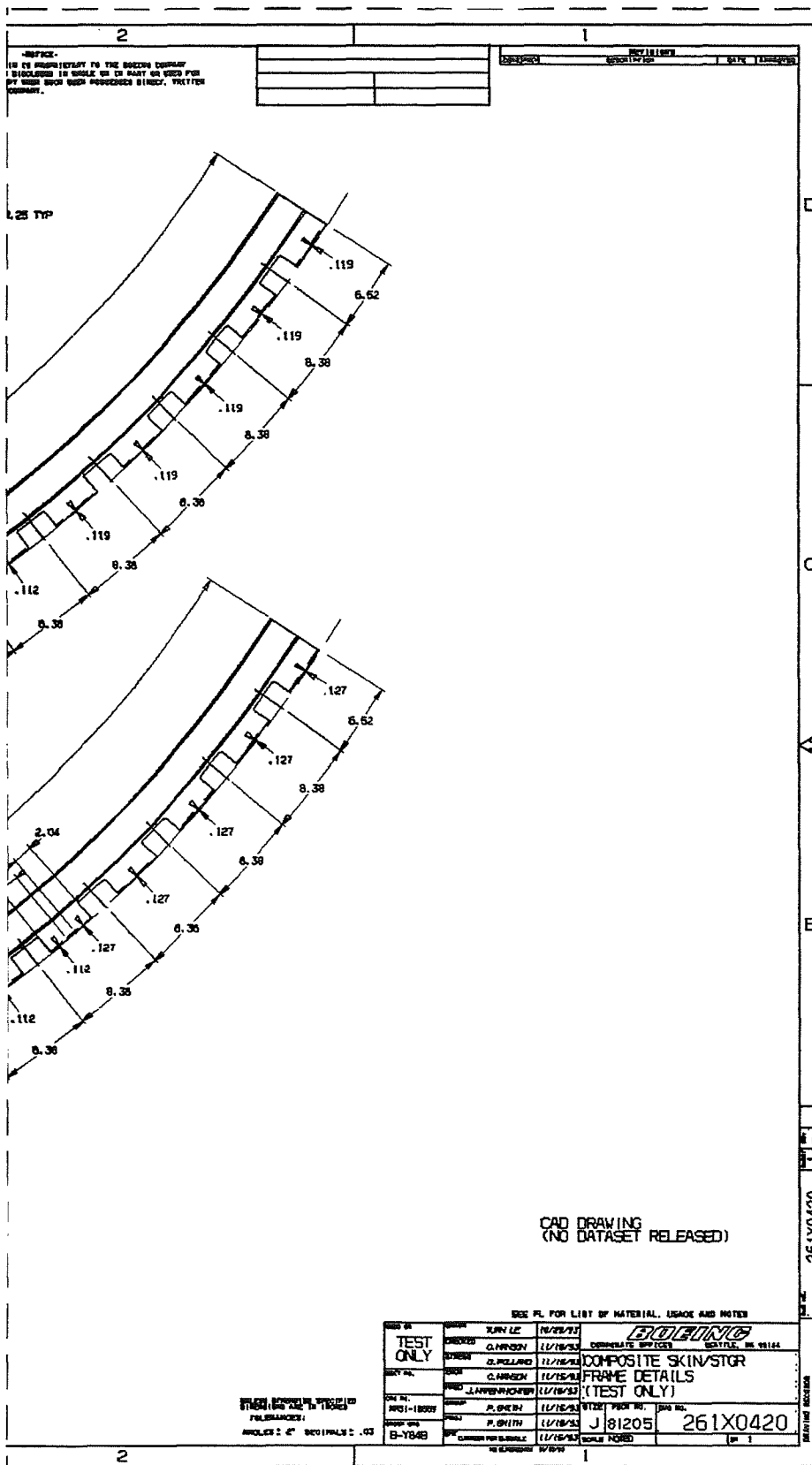
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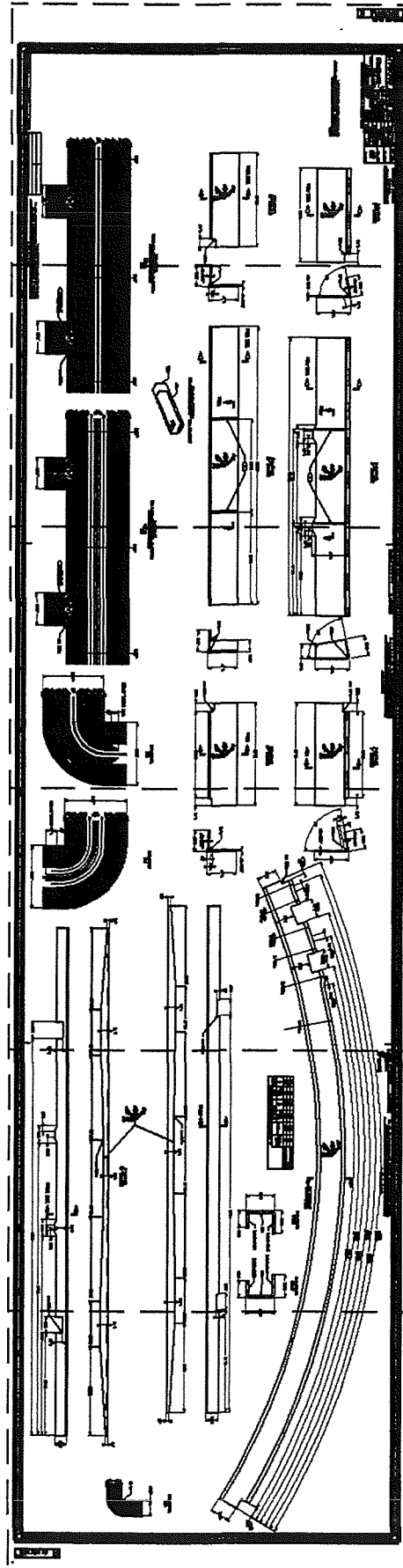
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B-33



B-34

B-35

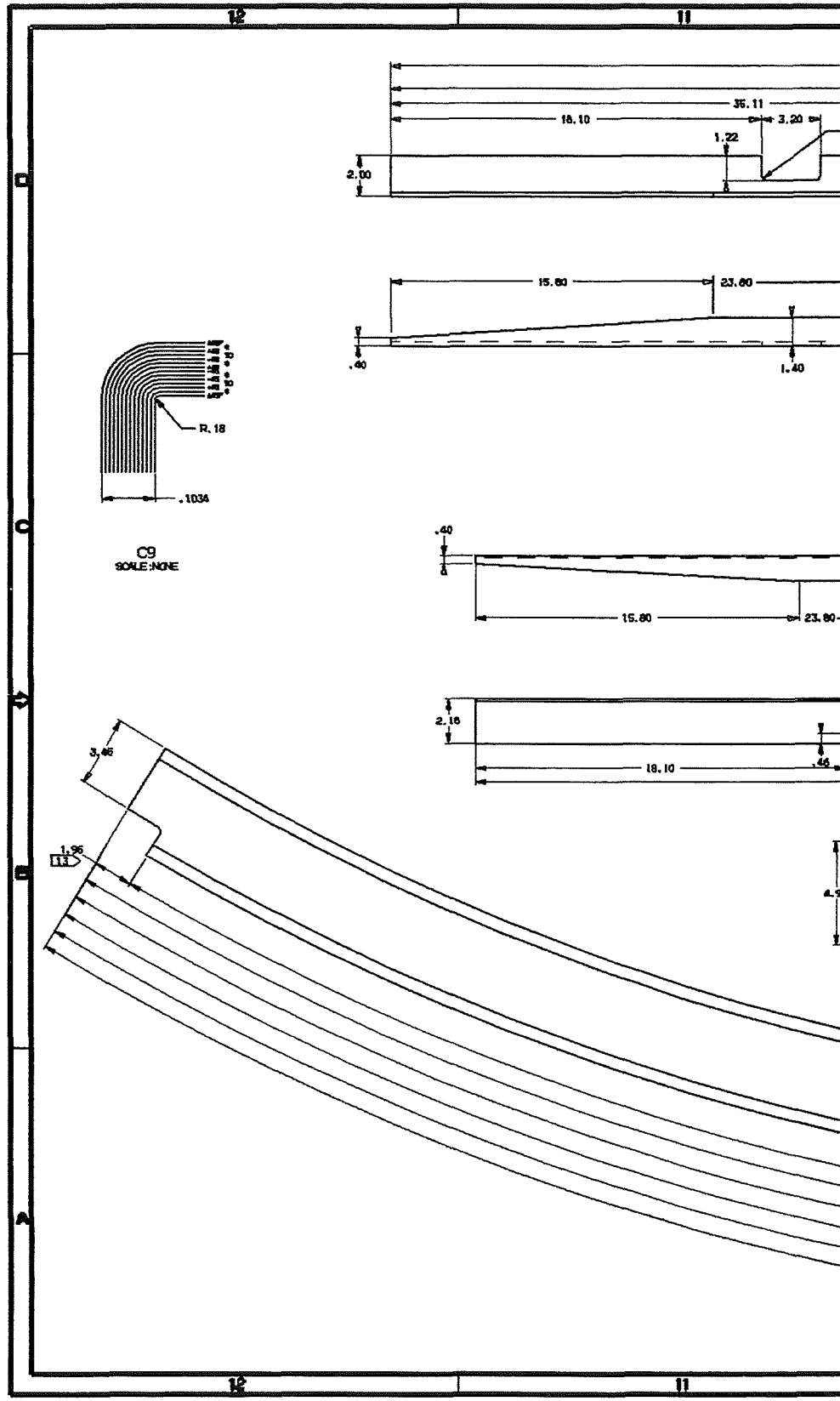
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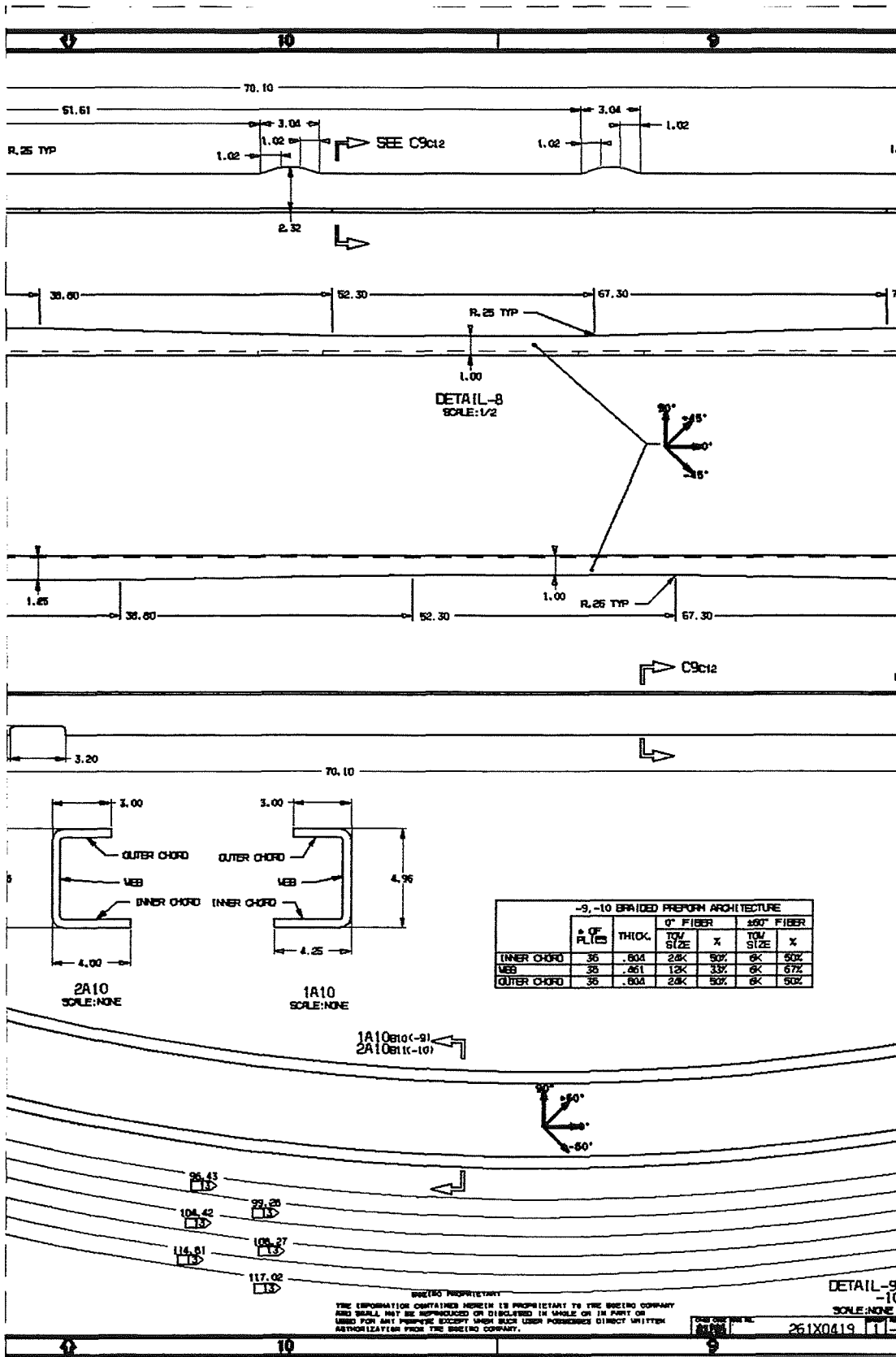
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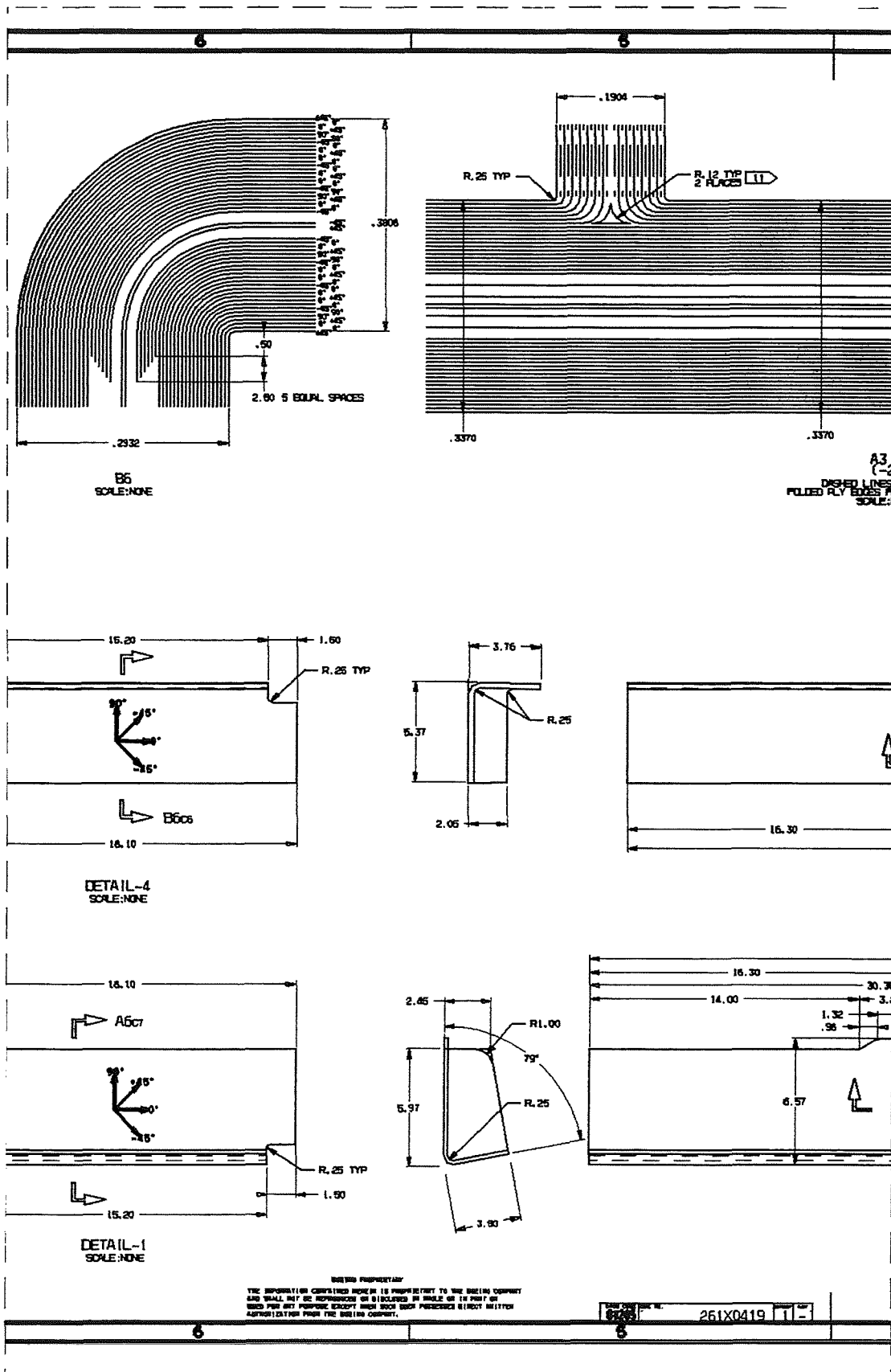
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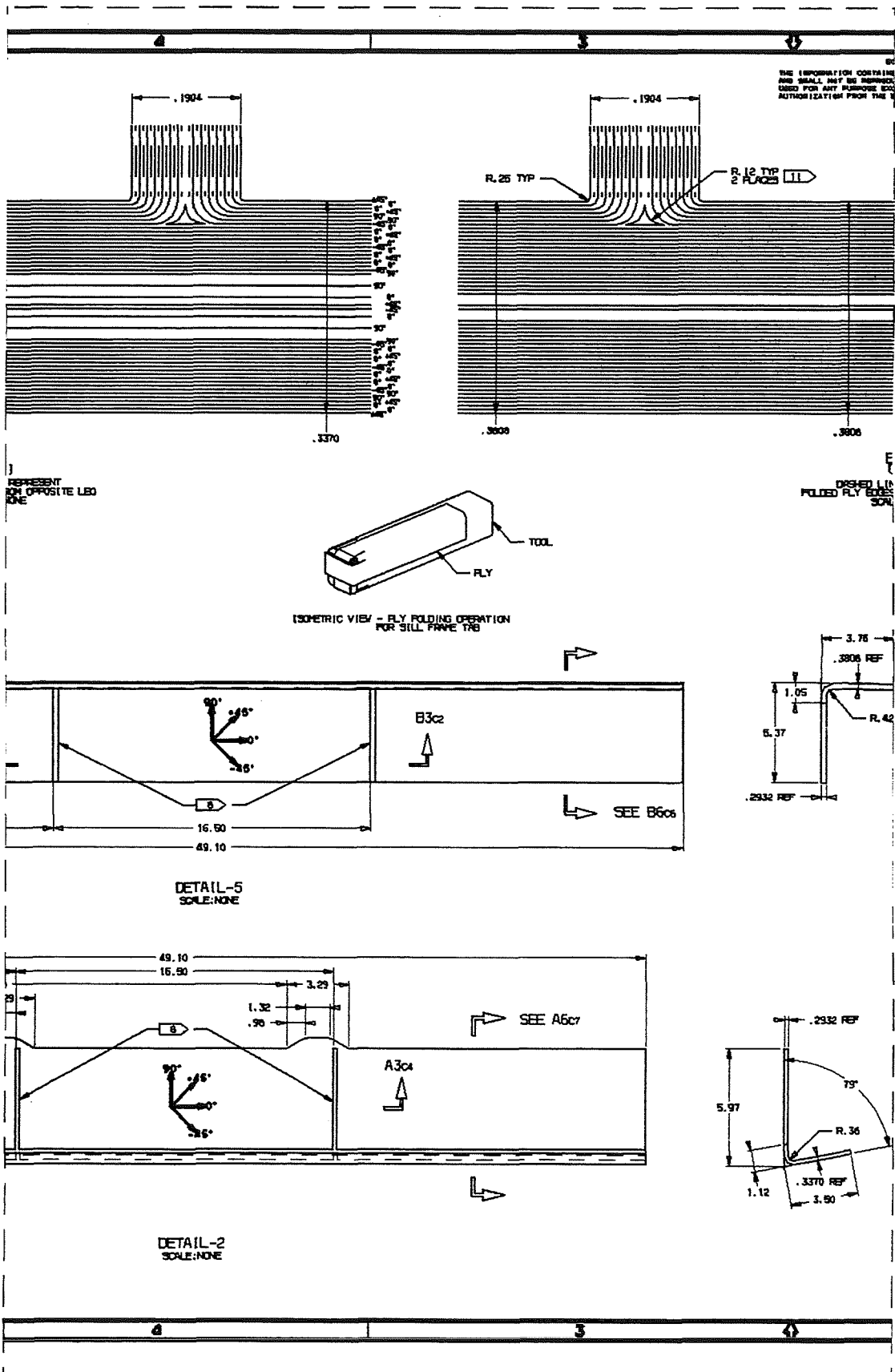
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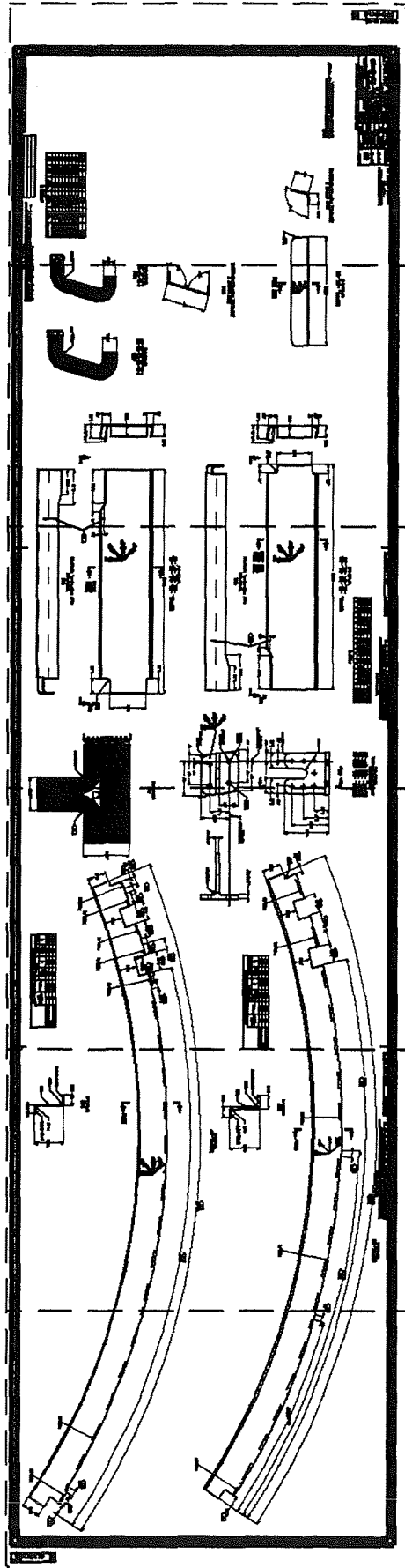












B-40

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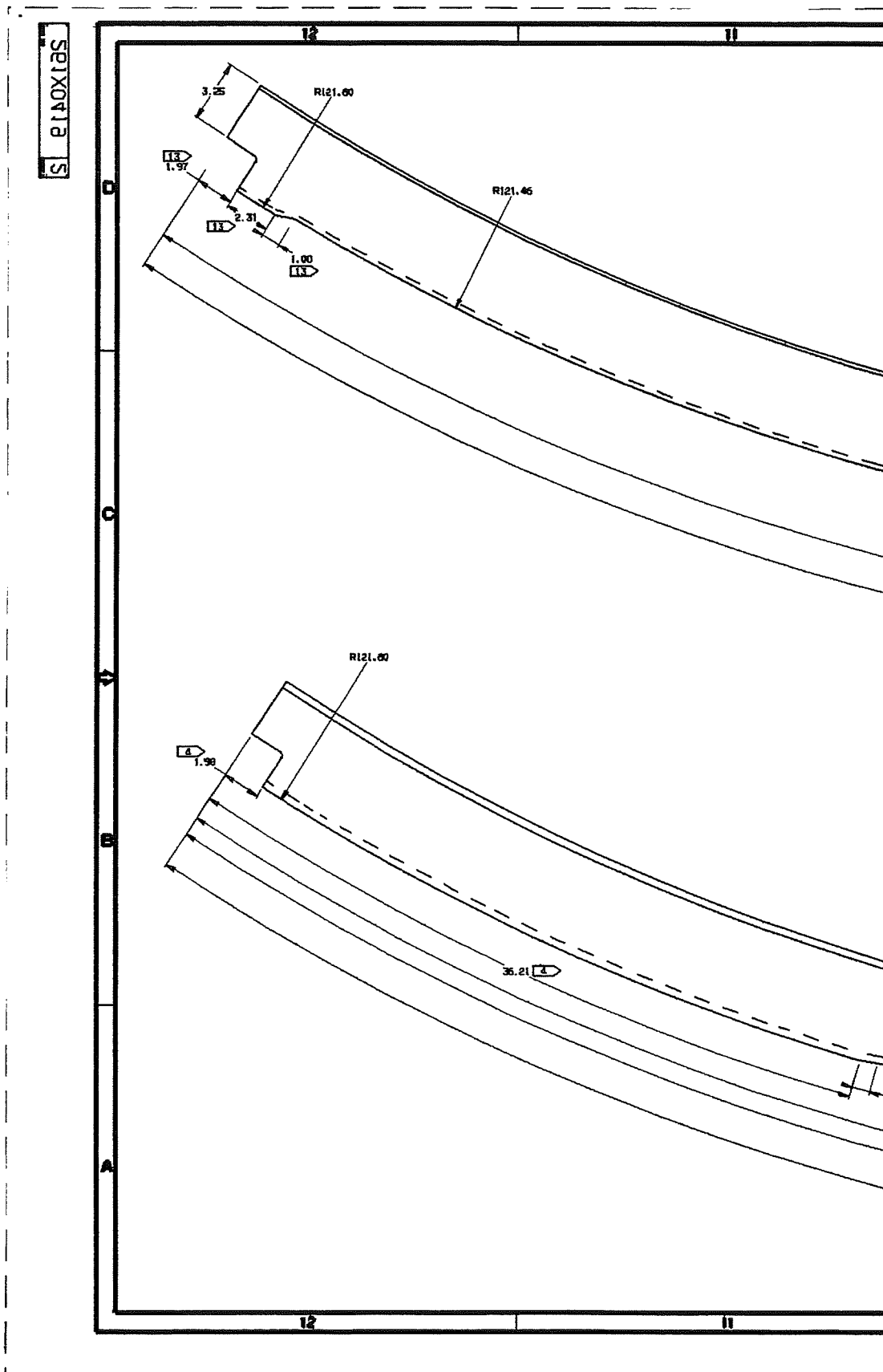
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B-43

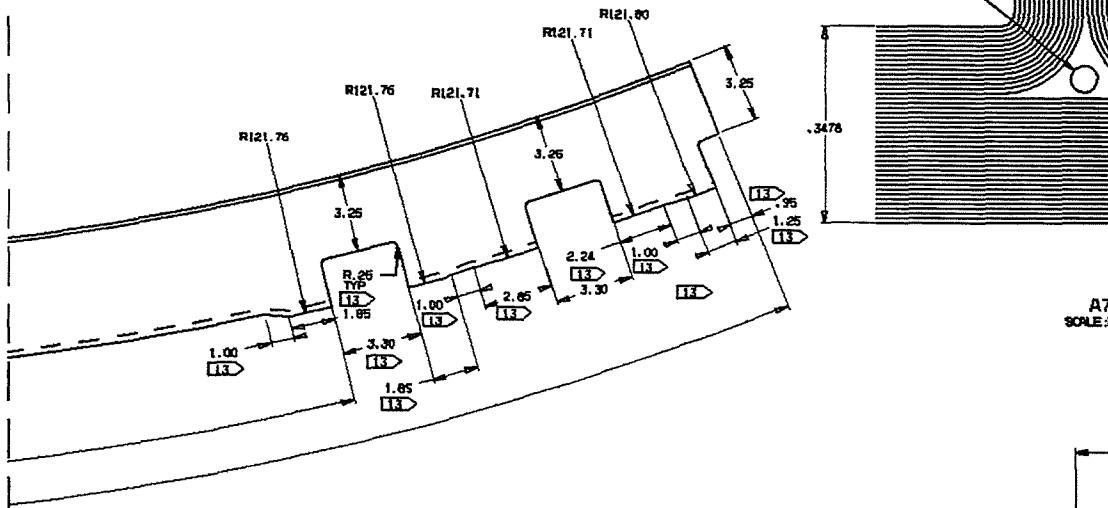
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B-45

B-46

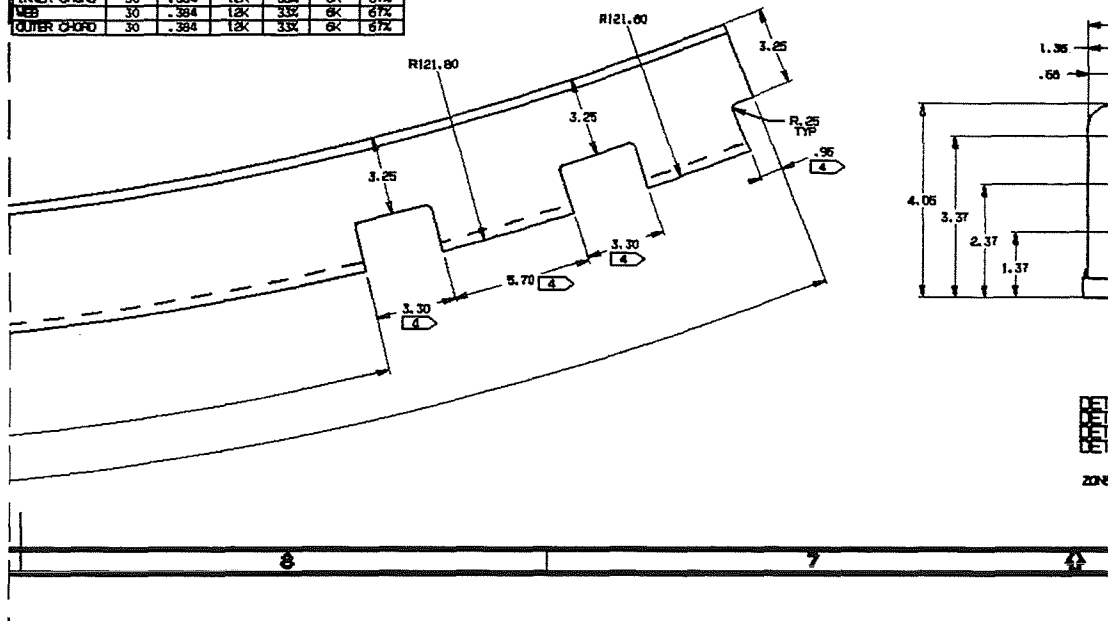


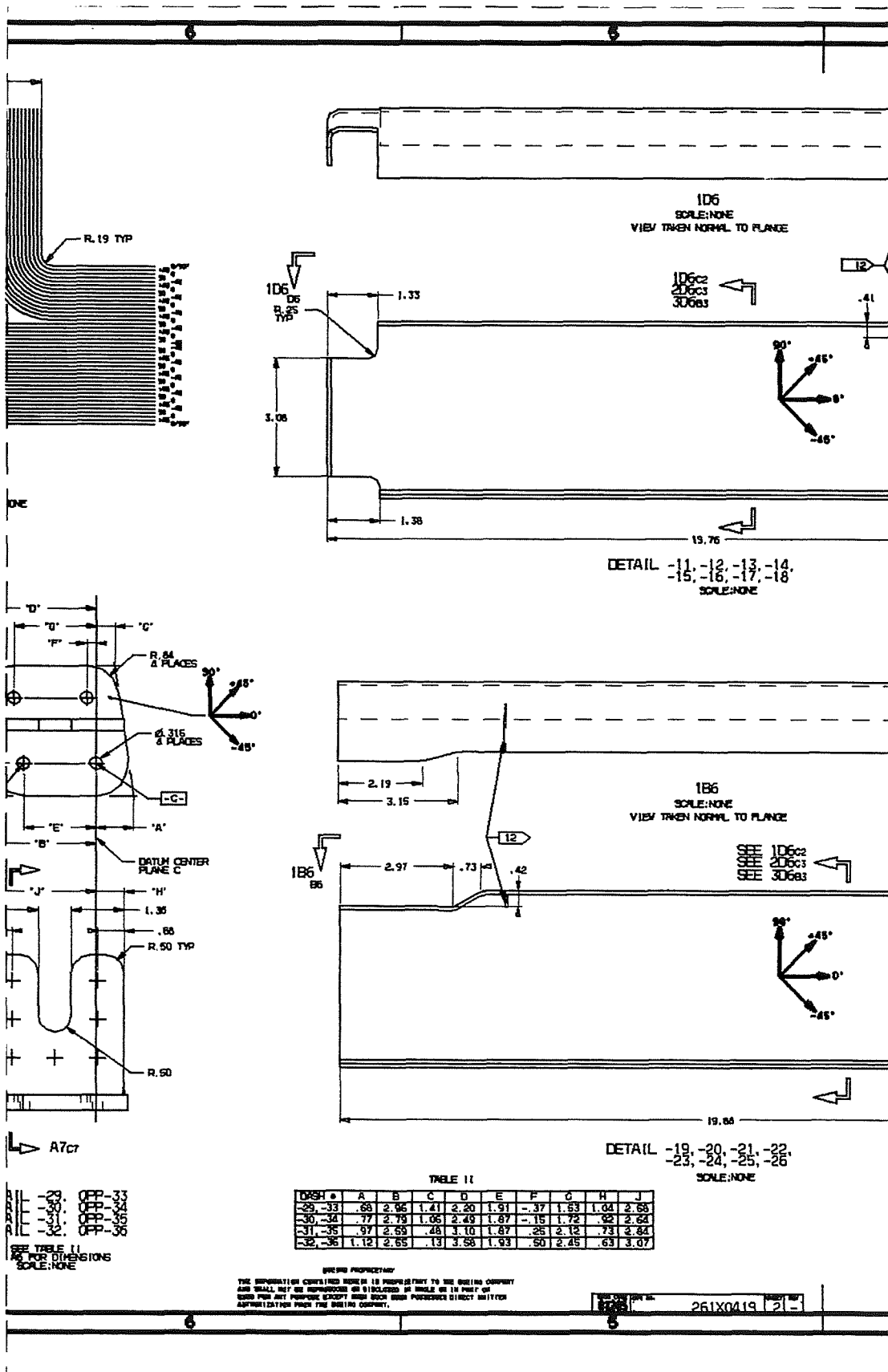
-37 BRAIDED PREFORM ARCHITECTURE						
	# OF PLIES	THICK.	0° FIBER		±60° FIBER	
			TOV SIZE	%	TOV SIZE	%
INNER CHORD	15	.192	12K	33%	6K	67%
WEB	15	.192	12K	33%	6K	67%
OUTER CHORD	15	.251	24K	50%	6K	50%

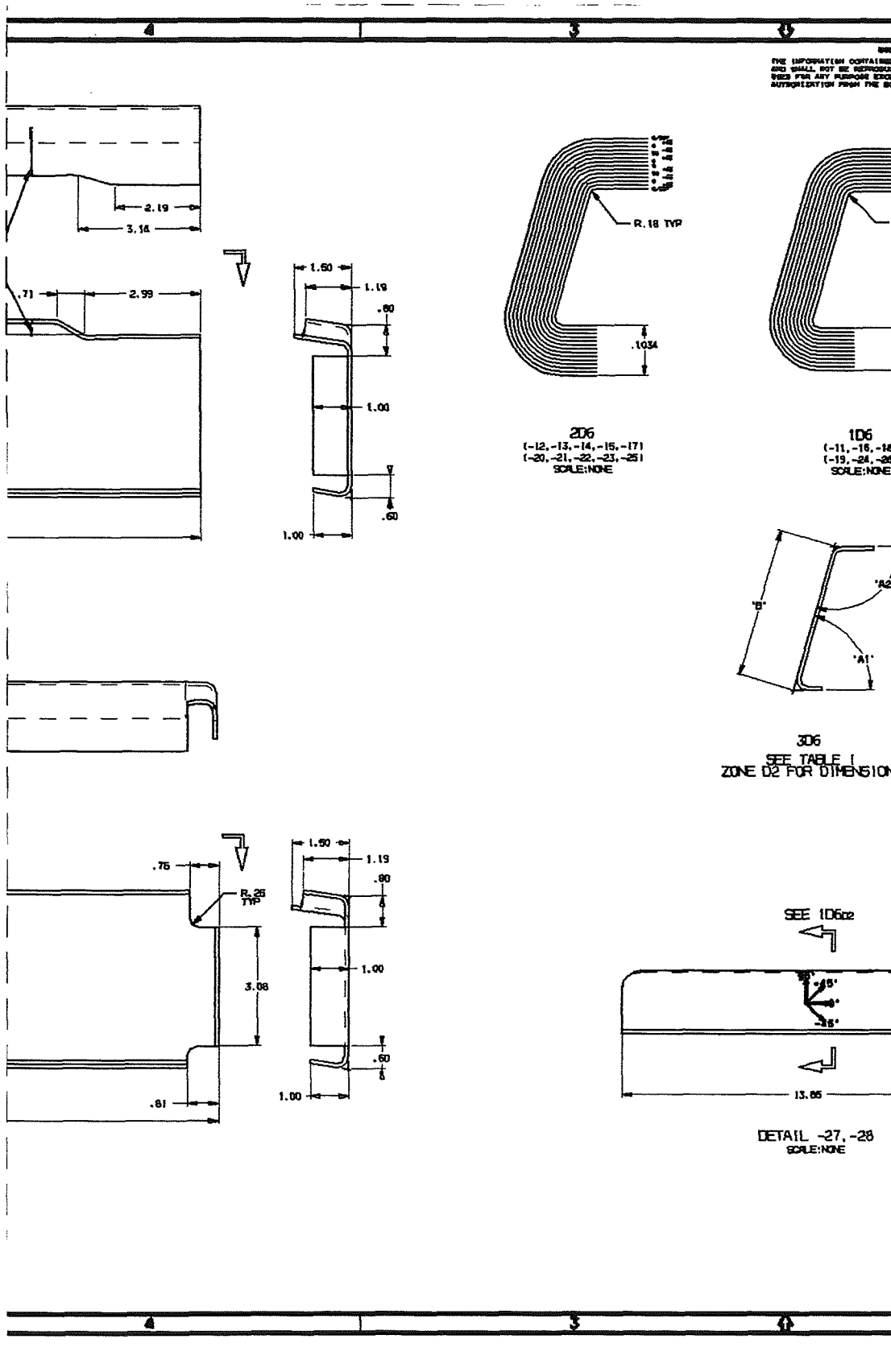


A7
SCALE 2X

-35 BRAIDED PREFORM ARCHITECTURE						
	# OF PLIES	THICK.	0° FIBER		±60° FIBER	
			TOV SIZE	%	TOV SIZE	%
INNER CHORD	30	.384	12K	33%	6K	67%
WEB	30	.384	12K	33%	6K	67%
OUTER CHORD	30	.384	12K	33%	6K	67%







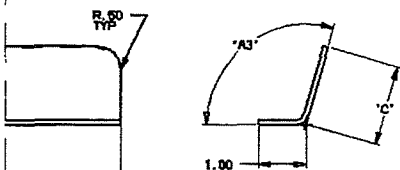
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BY WHOM SUCH UNDER POSSIBLE DIRECT WRITTEN
THE COMPANY.

[illegible]

DISH NO.	ANGLE A ₁	ANGLE A ₂	ANGLE A ₃	DIM B	DIM C
-11,-19	73.96°	106.04°	---	4.52	---
-12,-20	77.31°	102.33°	---	4.56	---
-13,-21	81.26°	98.74°	---	4.51	---
-14,-22	84.91°	98.09°	---	4.48	---
-15,-23	91.66°	88.14°	---	4.46	---
-16,-24	98.07°	81.33°	---	4.51	---
-17,-25	102.10°	77.90°	---	4.56	---
-18,-26	106.14°	73.86°	---	4.52	---
-27	---	---	94.65°	---	L.69
-30	---	---	67.74°	---	L.65

1. 18 TYP

re



SEE TABLE 1
ZONE D2 FOR DIMENSIONS

NOTE:
SOLIDS IN DATASET ARE FOR PICTORIAL
PURPOSE ONLY; THEY DO NOT ACCURATELY REFLECT
DIMENSIONS AS SHOWN ON Dwg SHEET.

UNLESS OTHERWISE SPECIFIED
DIMENSIONS ARE IN INCHES
TOLERANCES:
ANGLES & DECIMALS: .01

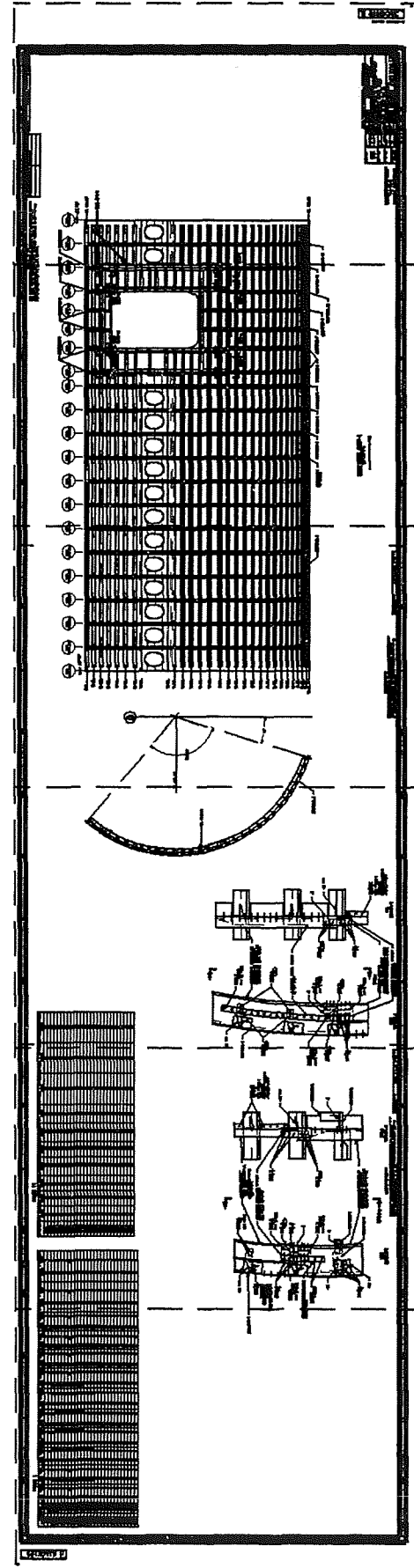
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C. HARRISON FOR J. HARRINGTON-STER 10/10/53

6170419

DATE OF BIRTH:

5140X125



B-47

B-48

B-49

B-50

B-51

B-52

B-53

1 8140X125

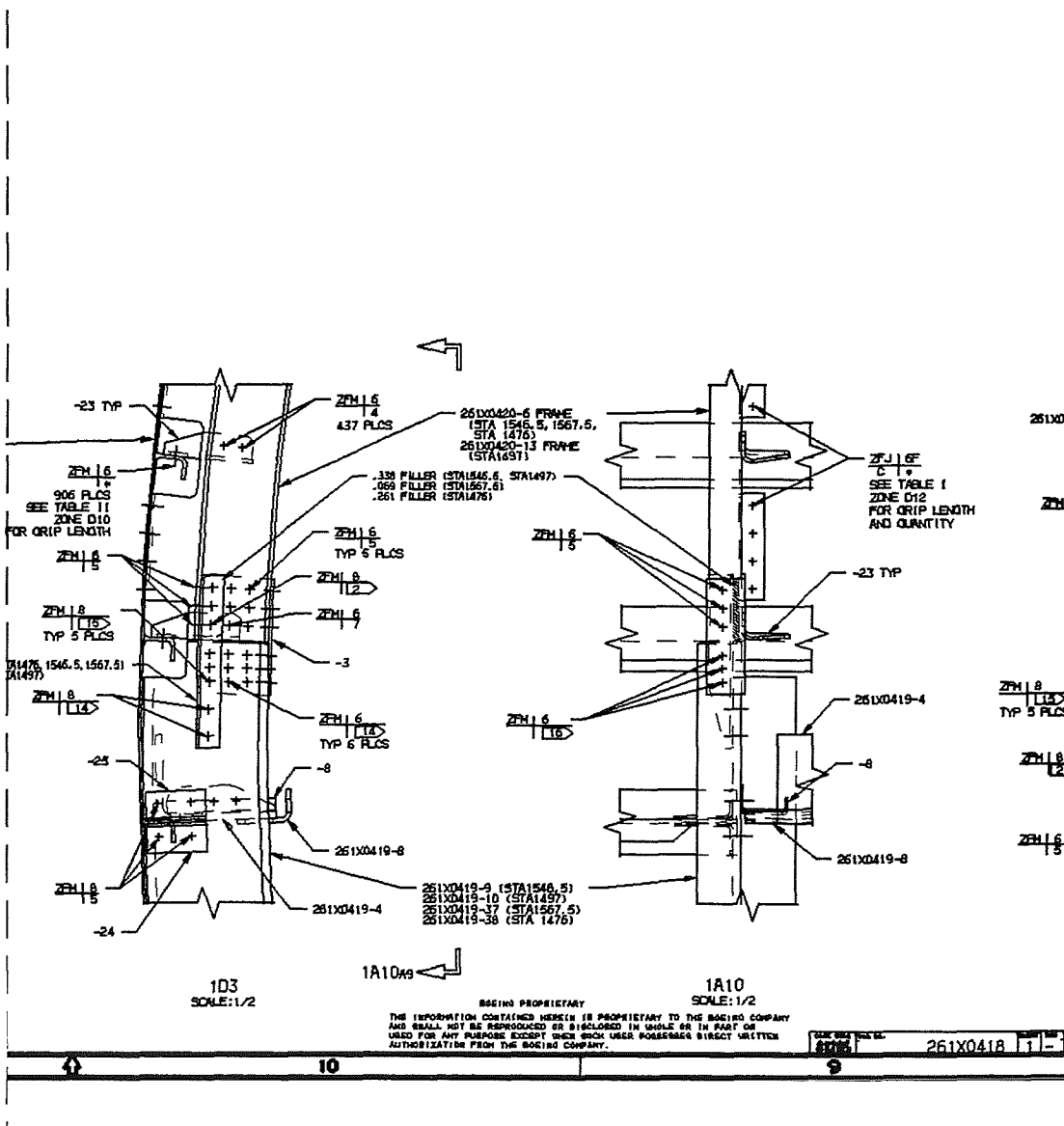
TABLE I

LOCATION BETWEEN STOR	STA 1455 GRIP	STA 1476 GRIP	STA 1497 GRIP	STA 1513.8 GRIP	STA 1530 GRIP	STA 1546.5 GRIP	STA 1567.5 GRIP	STA 1600 GRIP	STA 1601 GRIP	STA 1622.5 GRIP
S-9 - S-9	-6 4	-6 4	-6 4	-6 4	-6 4	-6 4	-6 4	-6 4	-6 4	-6 4
S-9 - S-10	-6 5	-6 5	-6 5	-6 5	-6 5	-6 5	-6 5	-6 5	-6 5	-6 5
S-10 - S-11	-6 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5
S-11 - S-12	-6 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5
S-12 - S-13	-6 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5
S-13 - S-14	-6 2	-10 2	-10 2	-10 2	-10 2	-10 2	-10 2	-10 2	-10 2	-10 2
S-14 - S-15	-6 2	-10 2	-10 2	-10 2	-10 2	-10 2	-10 2	-10 2	-10 2	-10 2
S-15 - S-16	-7 21	-11 21	-11 21	-11 21	-11 21	-11 21	-11 21	-11 21	-11 21	-11 21
S-16 - S-17	-6 2	-10 2	-10 2	-10 2	-10 2	-10 2	-10 2	-10 2	-10 2	-10 2
S-17 - S-18	-6 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5
S-18 - S-19	-6 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5
S-19 - S-20	-6 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5
S-20 - S-21	-6 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5
S-21 - S-22	-6 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5
S-22 - S-23	-6 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5
S-23 - S-24	-7 5	-11 5	-11 5	-11 5	-11 5	-11 5	-11 5	-11 5	-11 5	-11 5
S-24 - S-25	-6 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5
S-25 - S-26	-6 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5
S-26 - S-27	-6 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5	-10 5
S-27 - S-28	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5
S-28 - S-29	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5
S-29 - S-30	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5
S-30 - S-31	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5
S-31 - S-32	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5
S-32 - S-33	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5
S-33 - S-34	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5
S-34 - S-35	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5
S-35 - S-36	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5
S-36 - S-37	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5	-4 5
S-37 - S-38	-4 4	-4 4	-4 4	-4 4	-4 4	-4 4	-4 4	-4 4	-4 4	-4 4

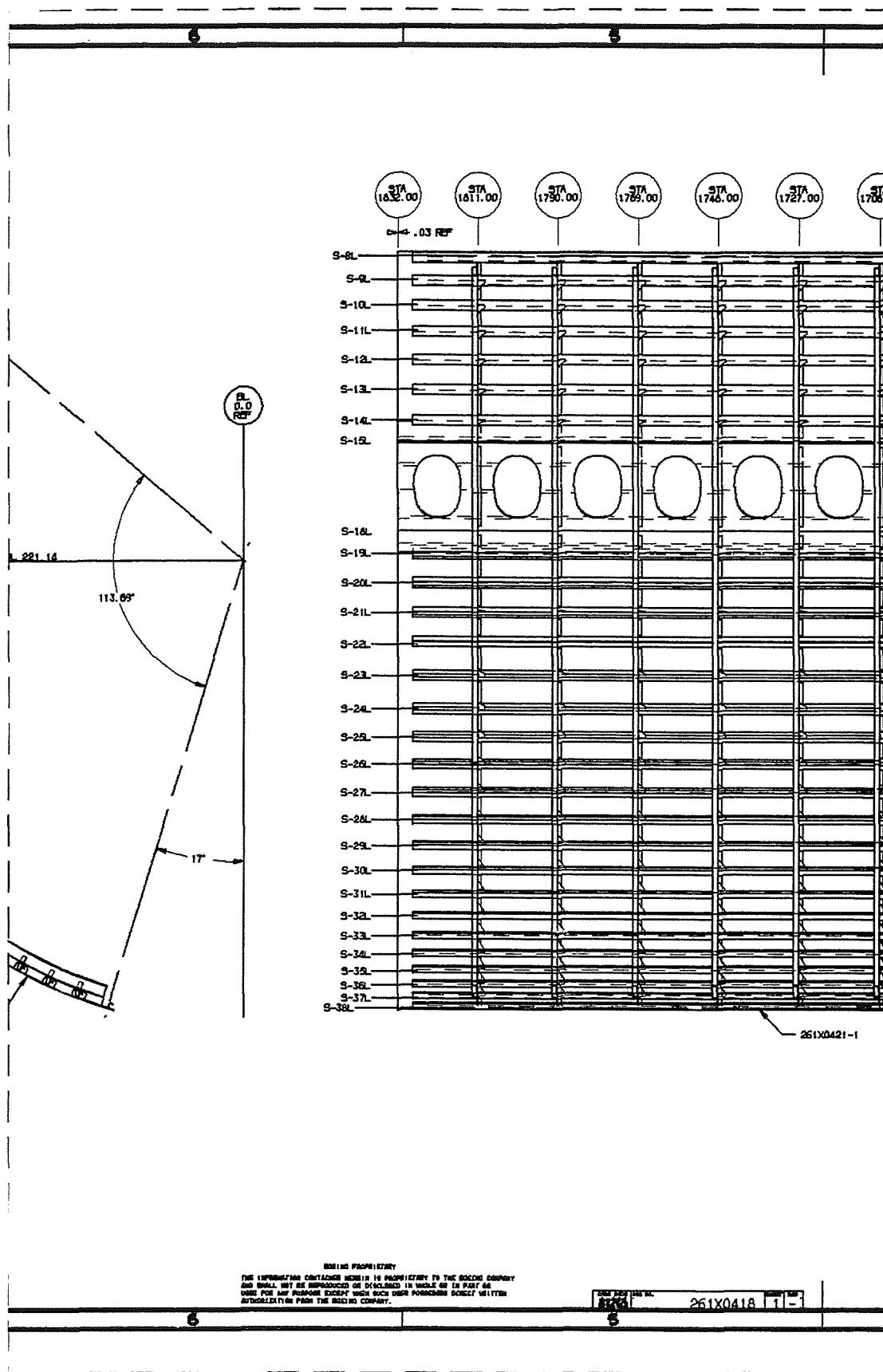
251X0421-1

251X0421-1

10				9											
TABLE II															
STA1643 - STA1727	STA1748 - STA1811	GRIP	QTY	STRINGER	STA 1455	STA 1476	STA 1497	STA 1613.6	STA 1630	STA 1646.6	STA 1667.6	STA1680 - STA1622	STA1643 - STA		
-4	4	-4	4	S-2	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-10	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-11	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-12	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-13	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-14	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-15	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-16	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-17	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-18	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-19	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-20	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-21	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-22	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-23	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-24	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-25	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-26	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-27	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-28	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-29	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-30	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-31	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-32	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-33	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-34	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-35	-4	-4	-4	-4	-4	-4	-4	-4	-4		
-4	5	-4	5	S-36	-4	-4	-4	-4	-4	-4	-4	-4	-4		
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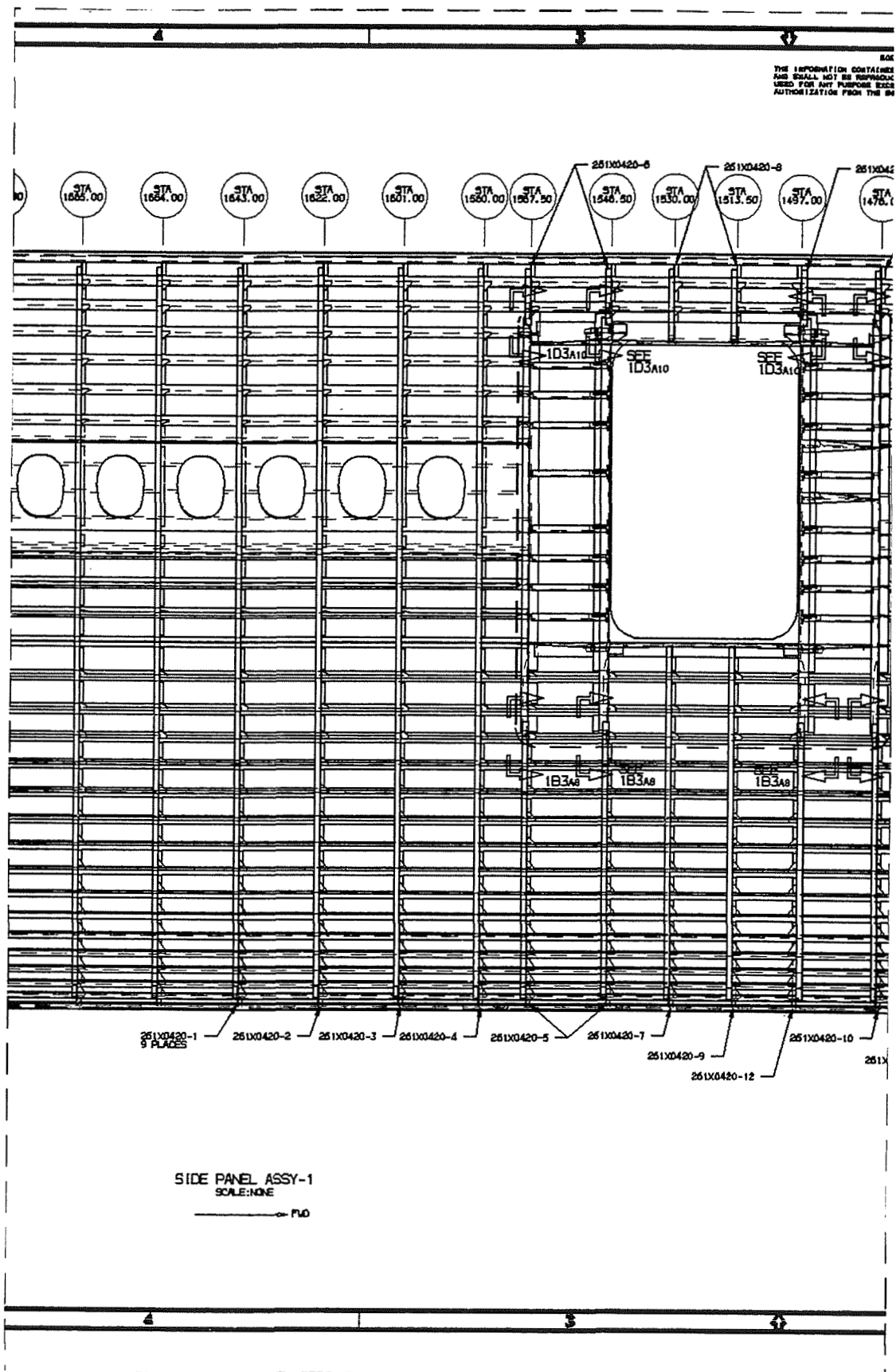




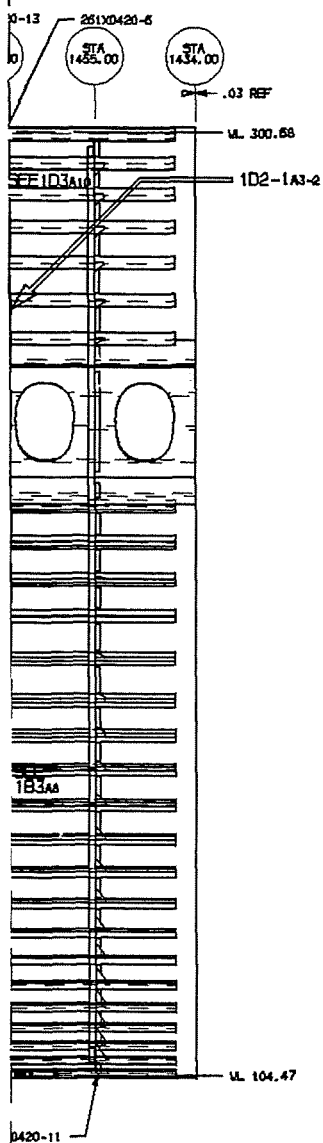


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261X0418 1-1



100-443886-100



0420-11

261X0418 11-5

SEE FL FOR LIST OF MATERIALS, USAGE AND NOTES			
TEST QM	GRON	TURN LE	12/16/83
TEST QM	SKETCHED	C. HANSEN	12/21/83
	STRESS	D. POLLAND	12/25/83
TEST NO.	ERROR	C. HANSEN	12/22/83
TEST NO.	OFF	C. HANSEN	12/22/83
	P. B. H. H.	C. HANSEN	12/22/83
GROUP NO.	GROUP NO.	P. B. H. H.	12/22/83
B-Y-042	C. HANSEN	C. HANSEN	12/22/83

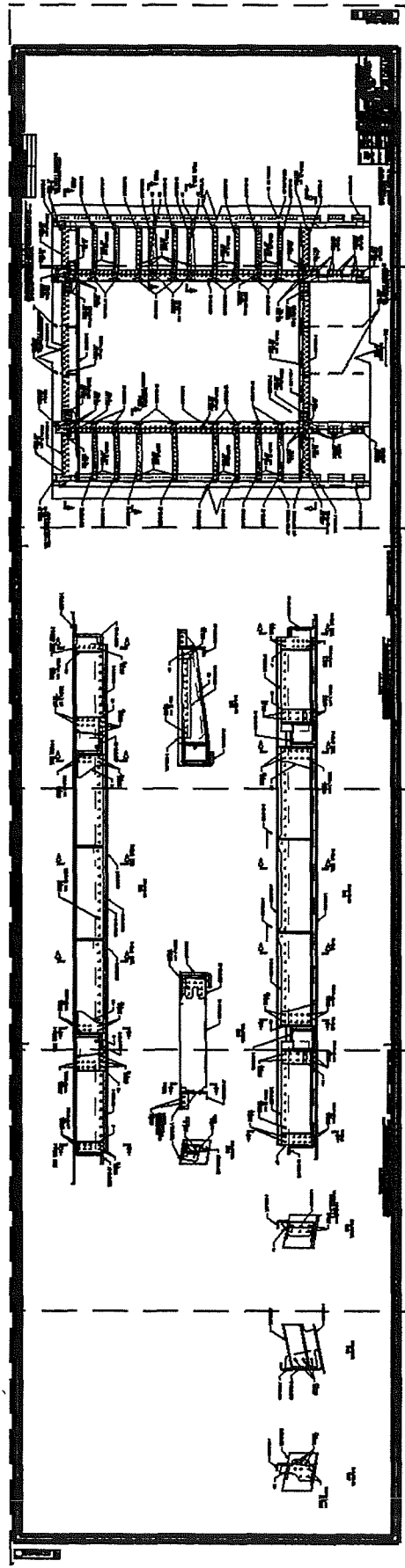
ALL DIMENSIONS UNLESS SPECIFIED
DIMENSIONS ARE IN INCHES
TOLERANCES:
ANGLES ± 1° DECIMALS ± .02

[illegible]

1 1047 3214 REV 7/91

8140X1a5

B-54



B-55

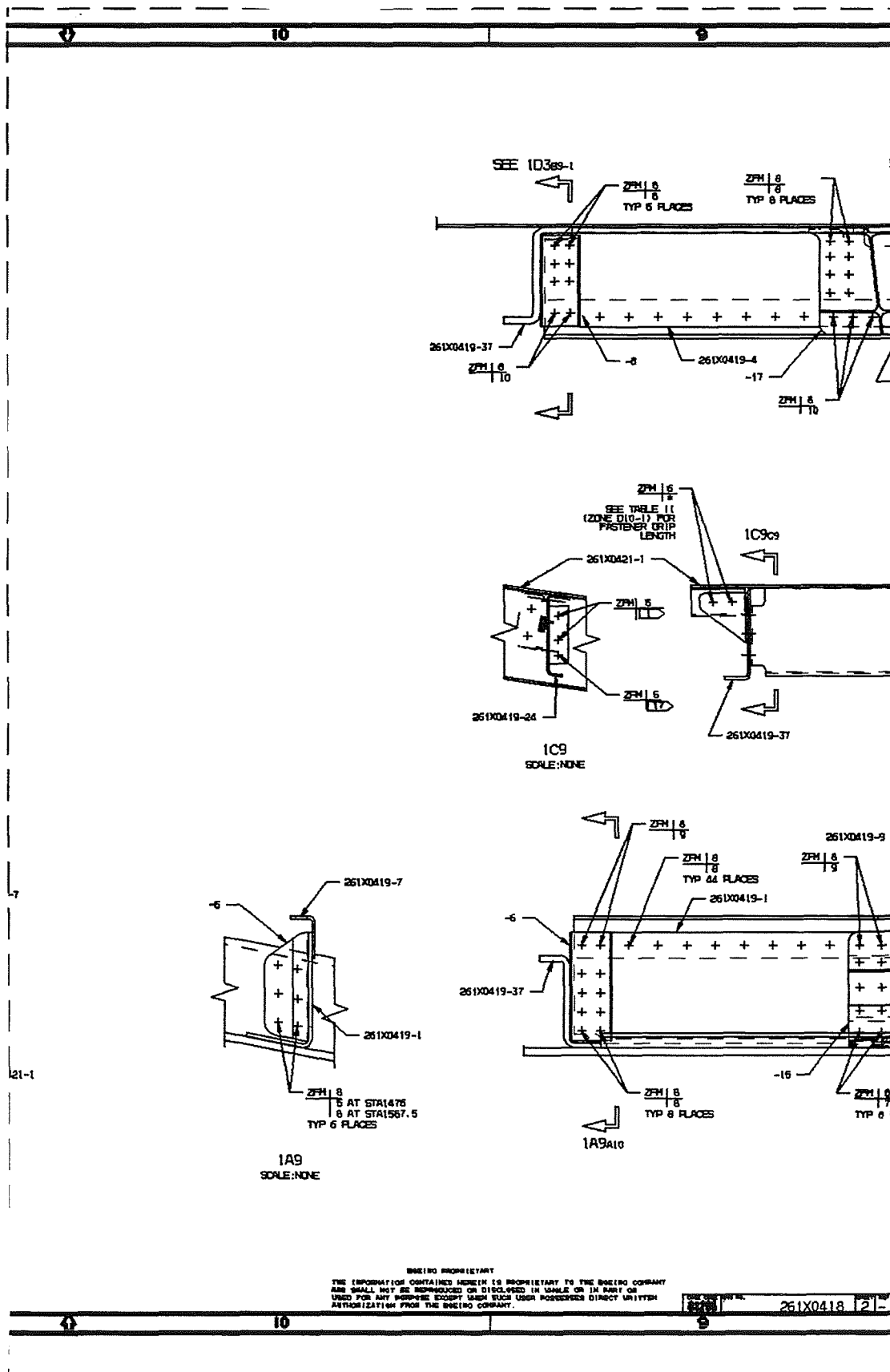
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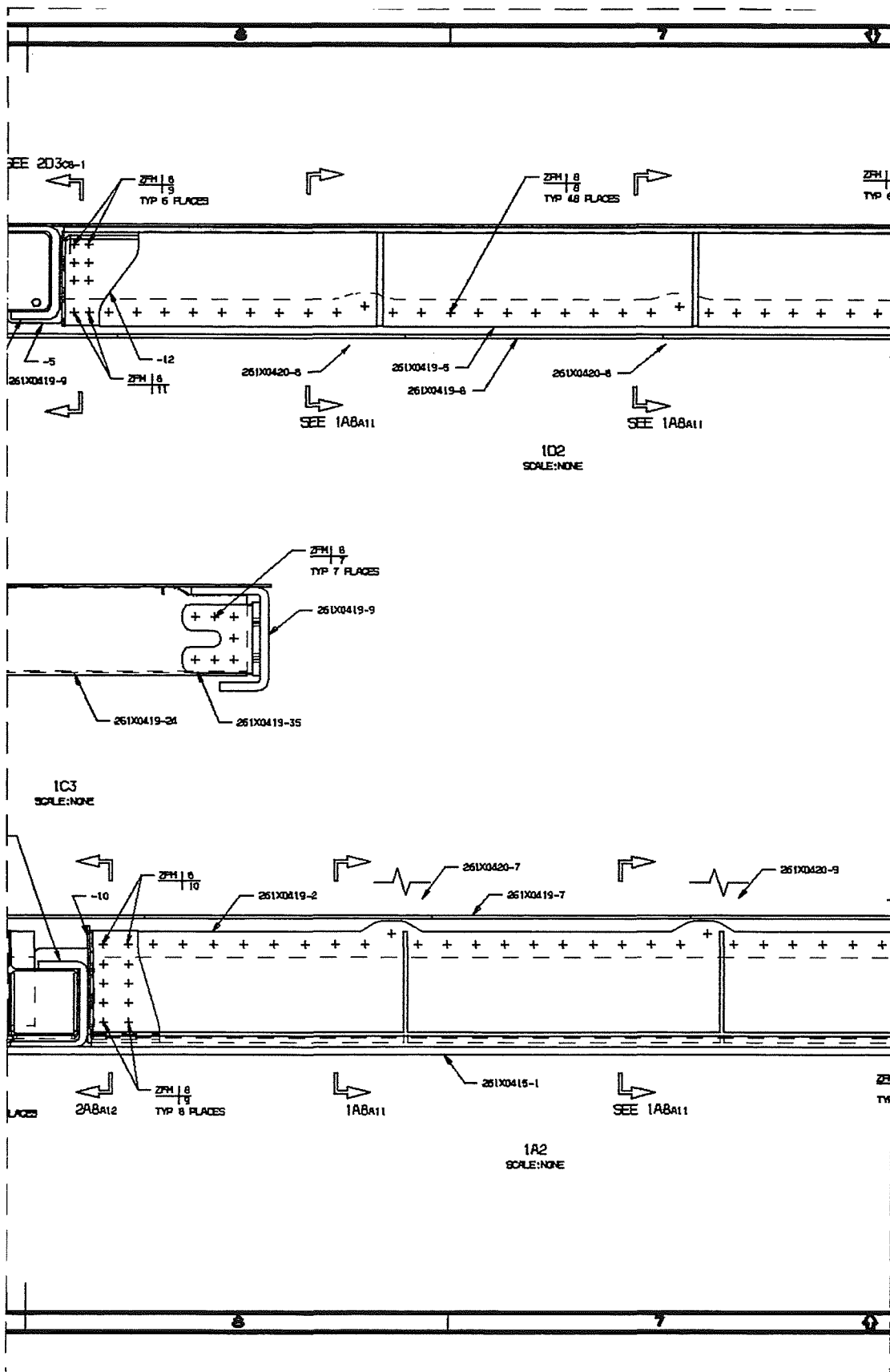
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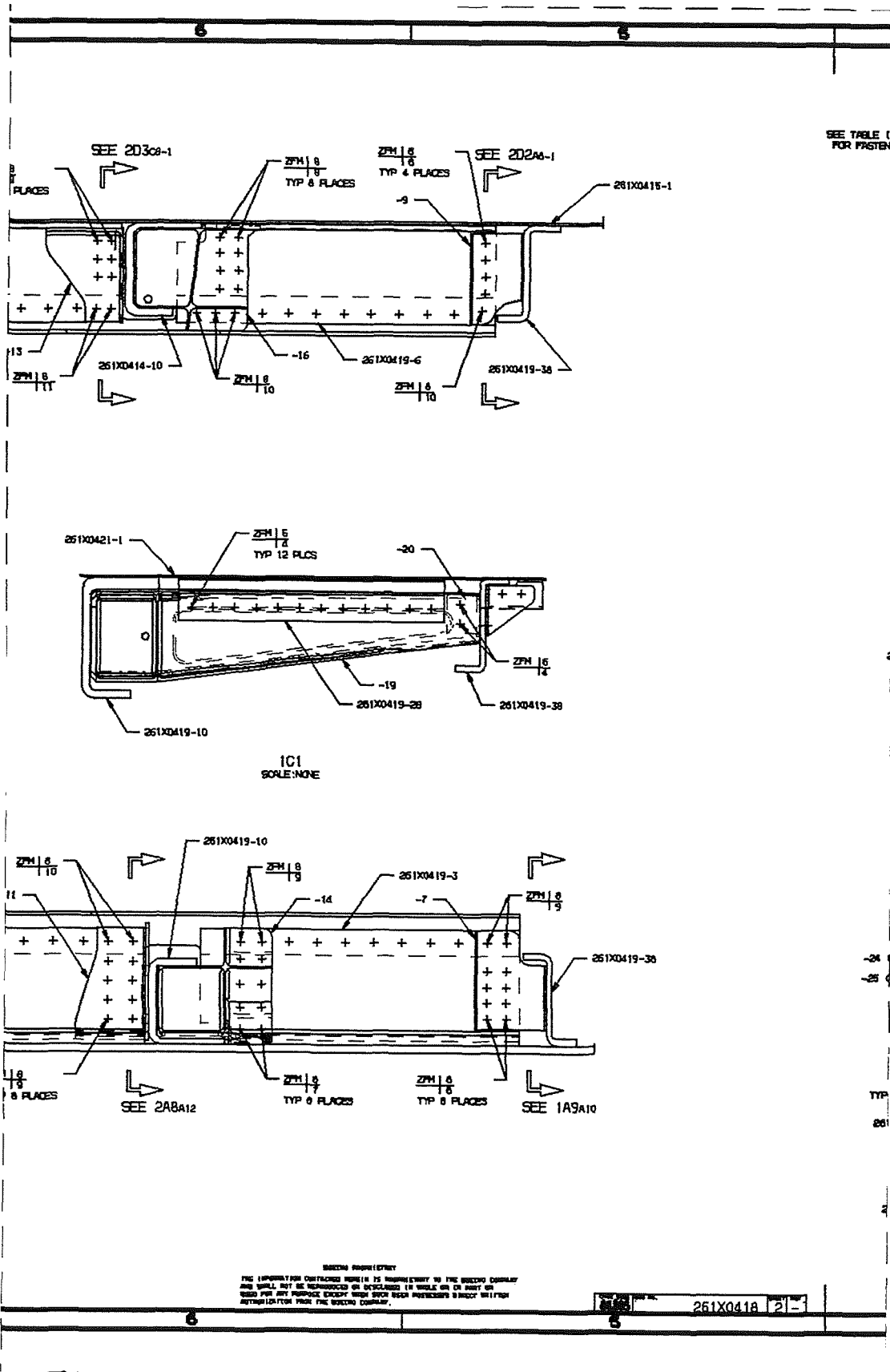
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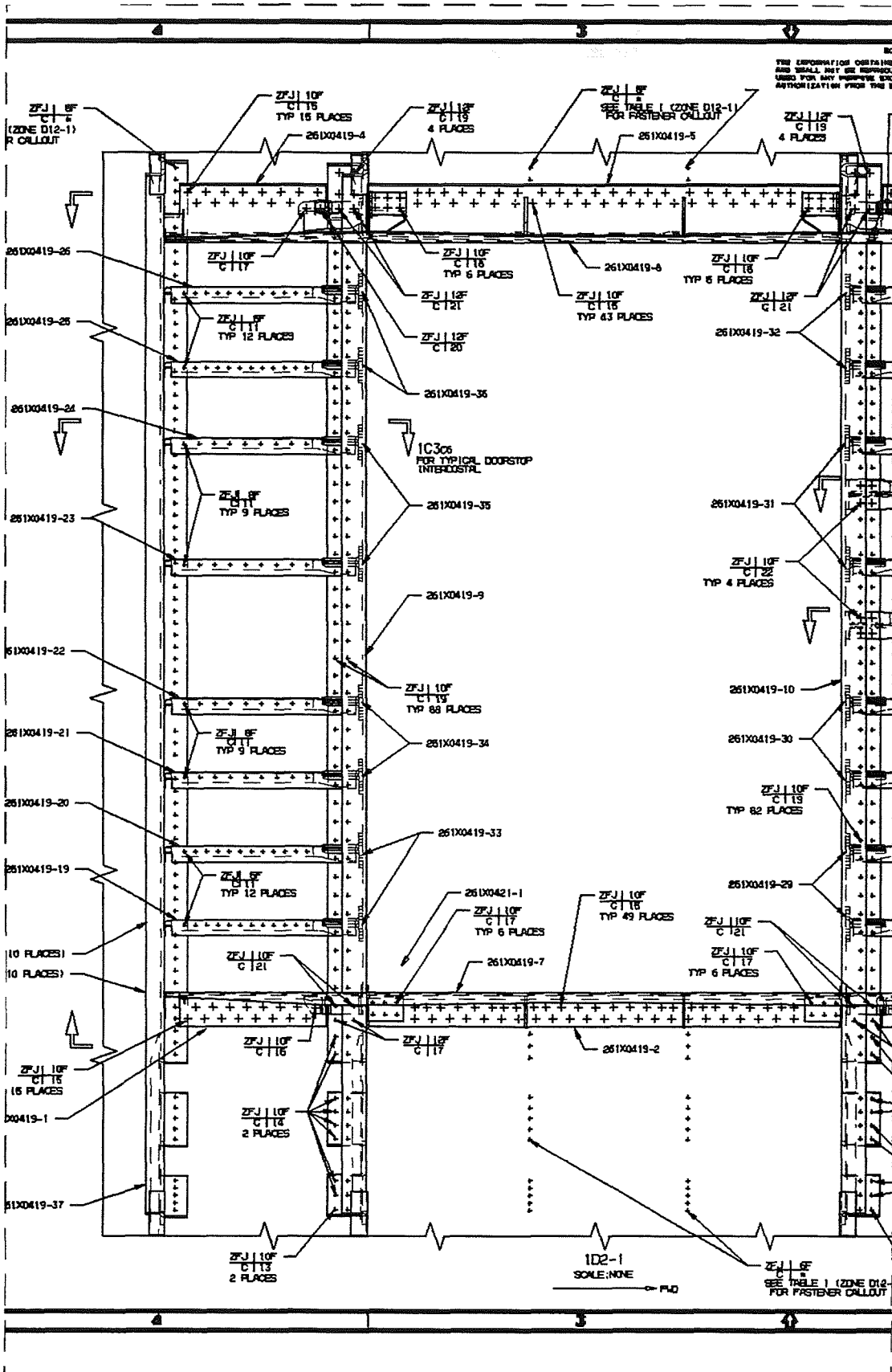
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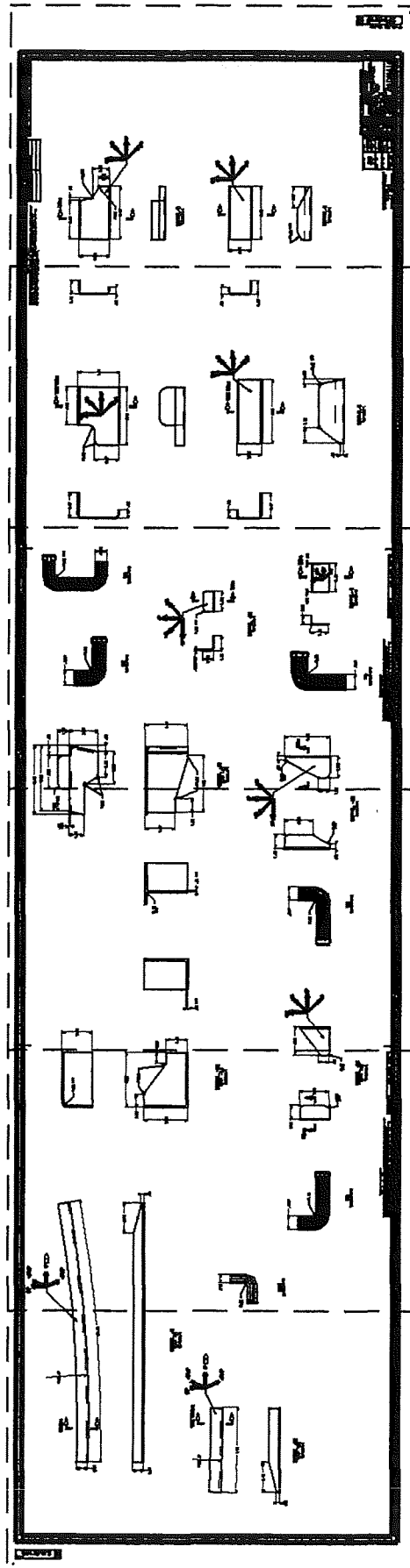












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B-67

B-66

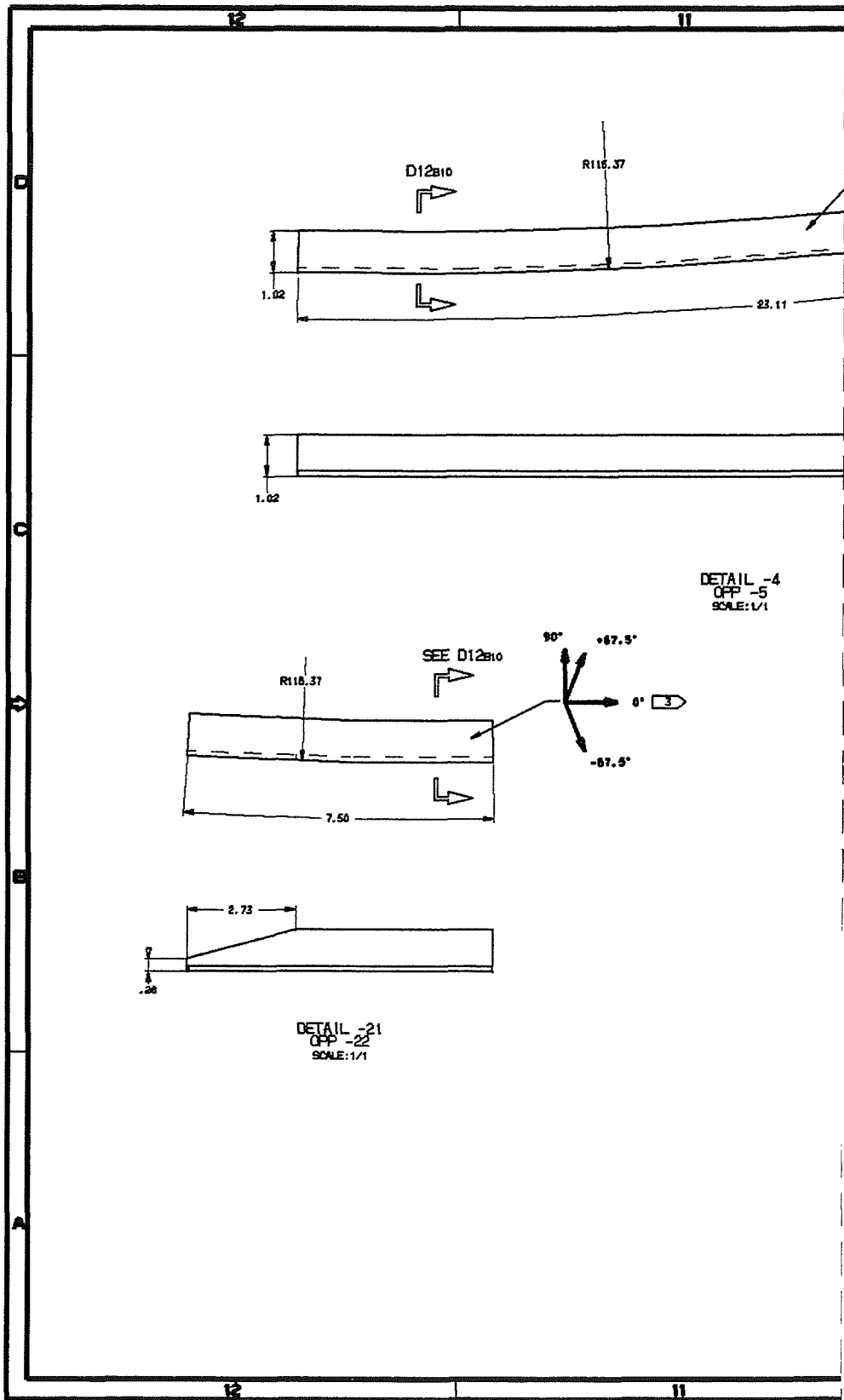
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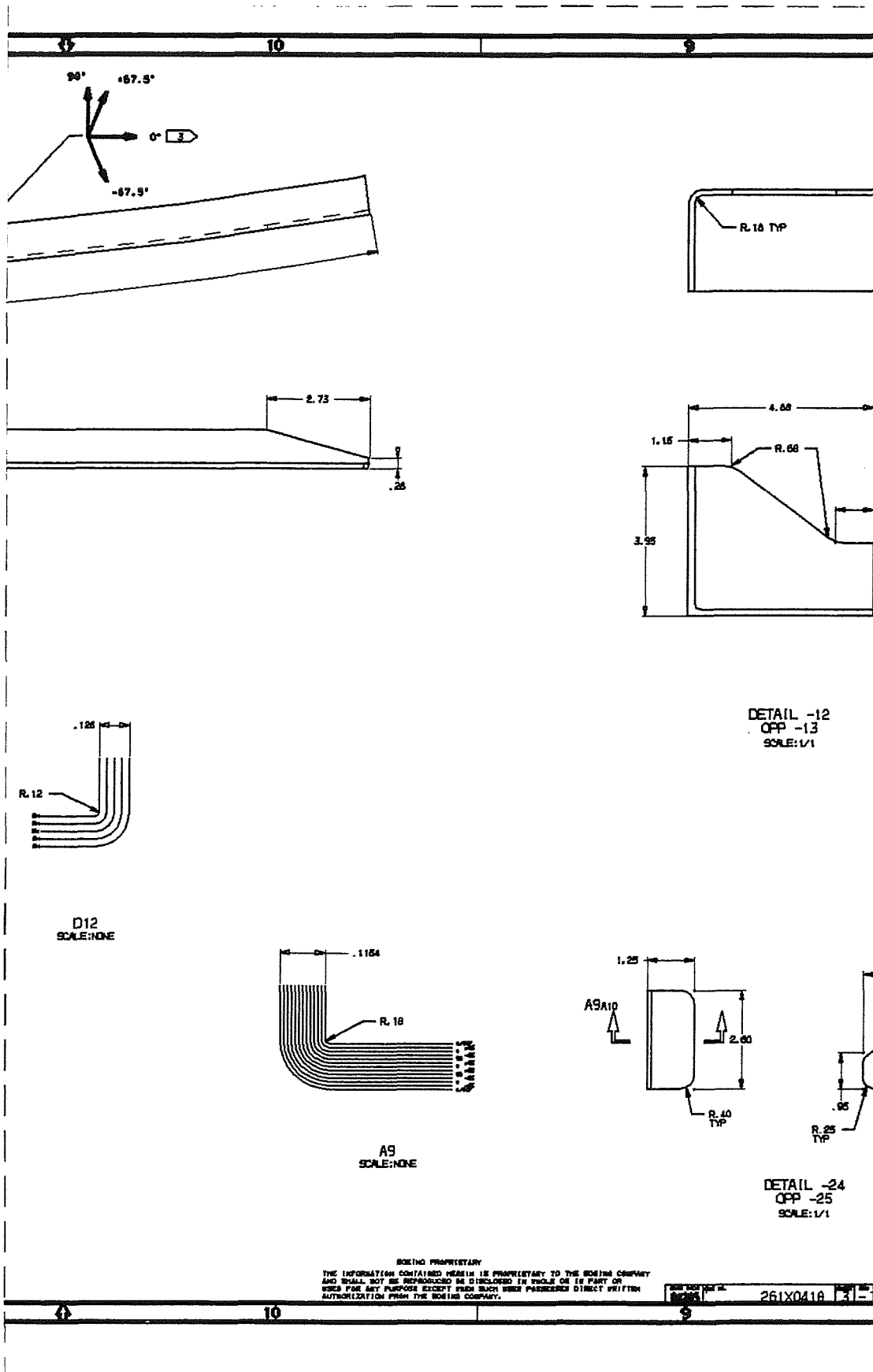
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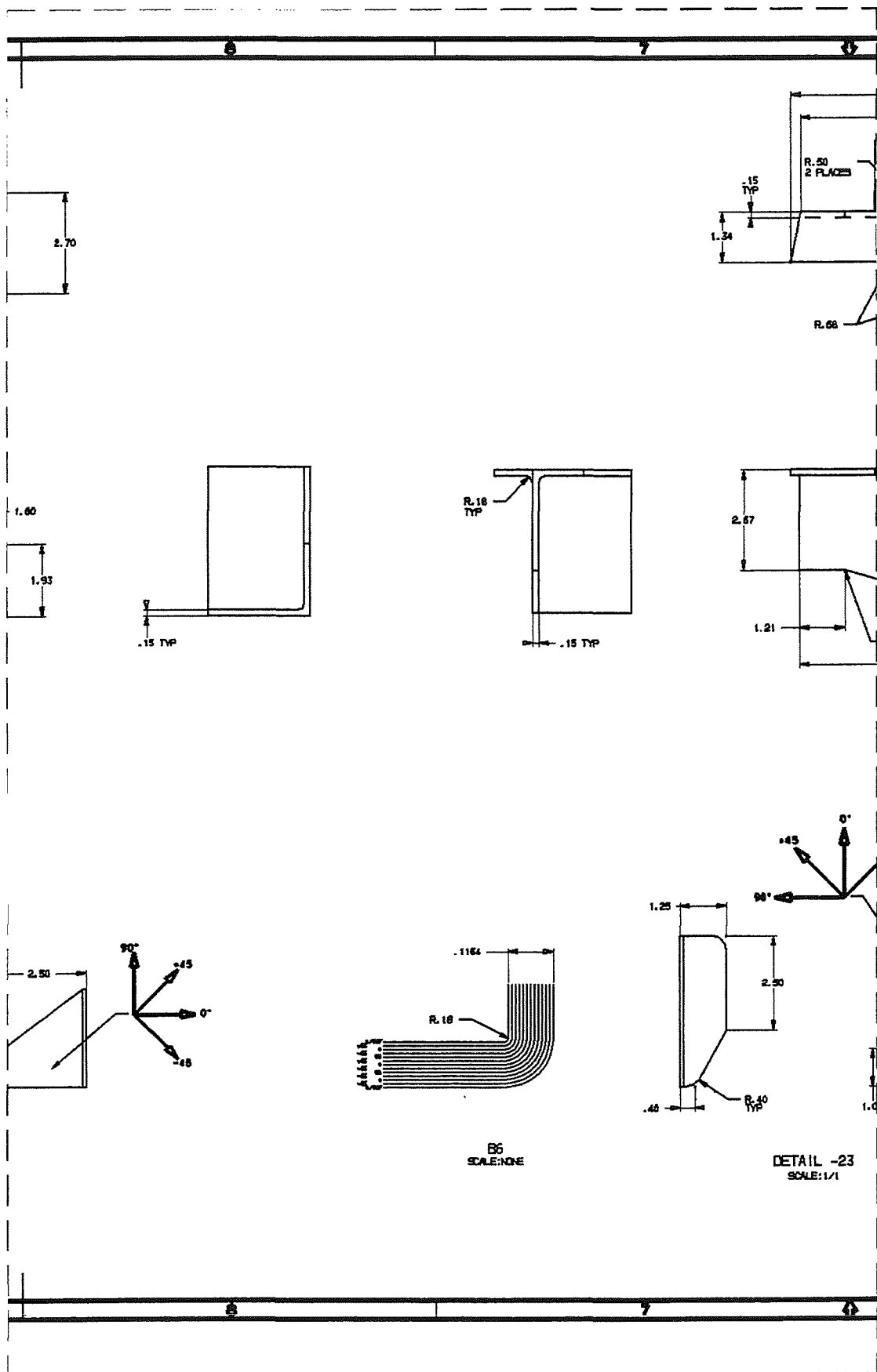
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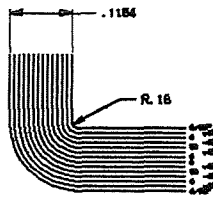
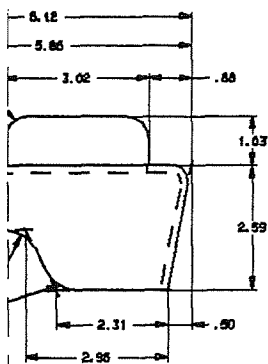
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501X0418 3

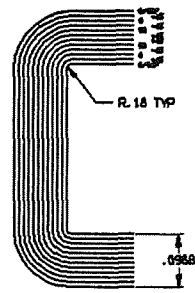




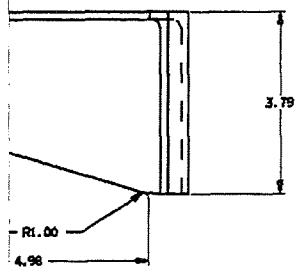




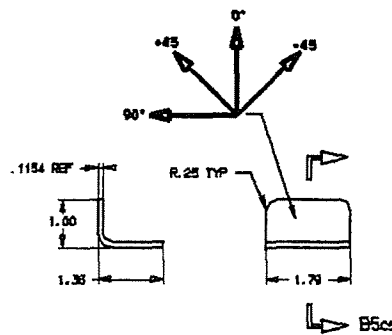
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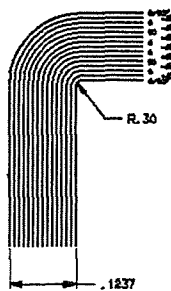
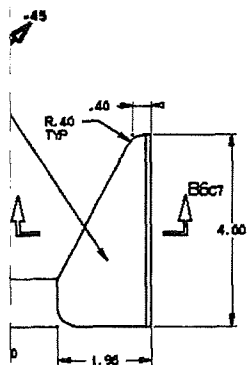
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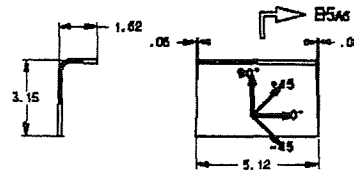
DETAIL -10
OPP -11
SCALE:1/1



DETAIL -20
SCALE:1/1



A6
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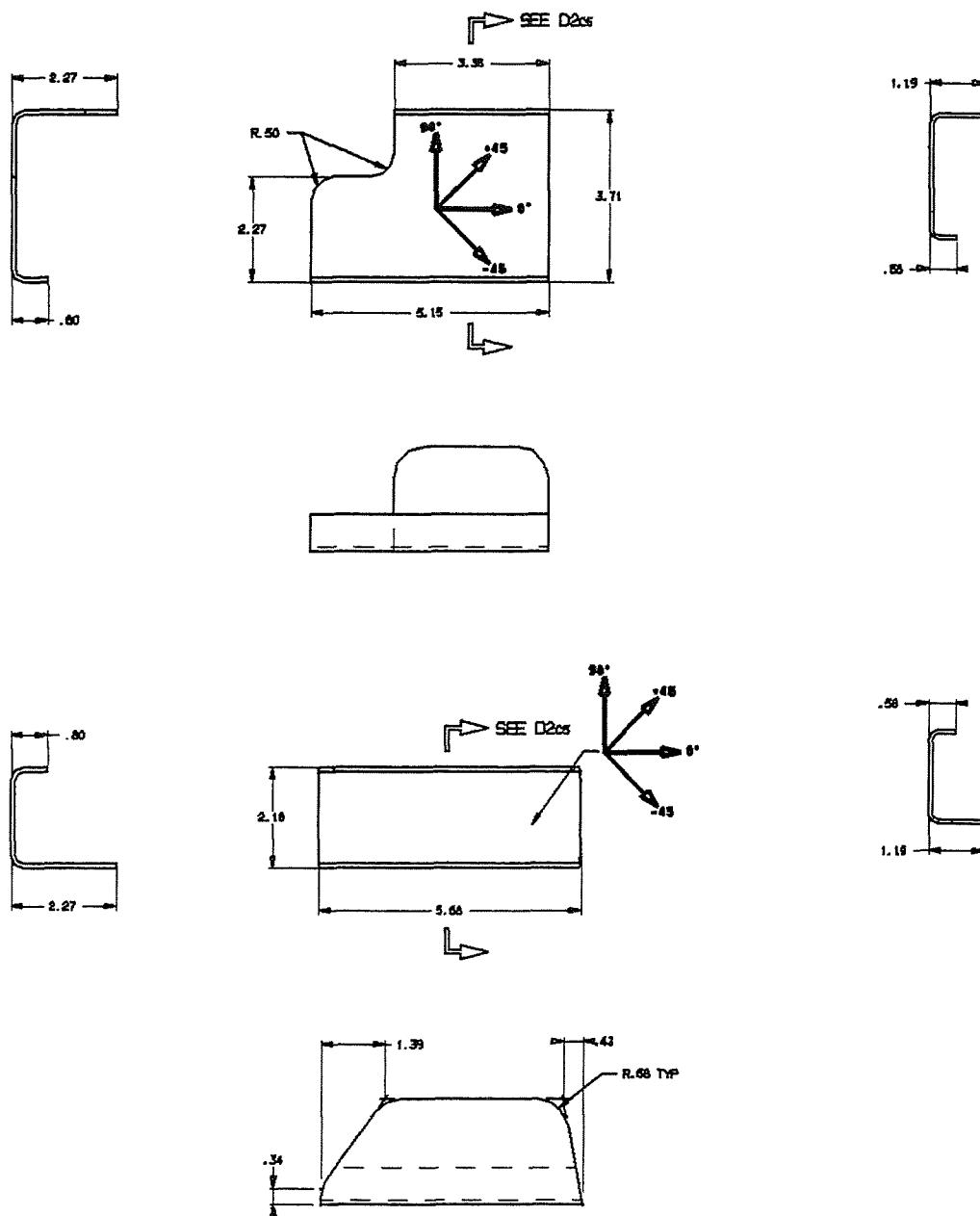


DETAIL-3
SCALE:1/2

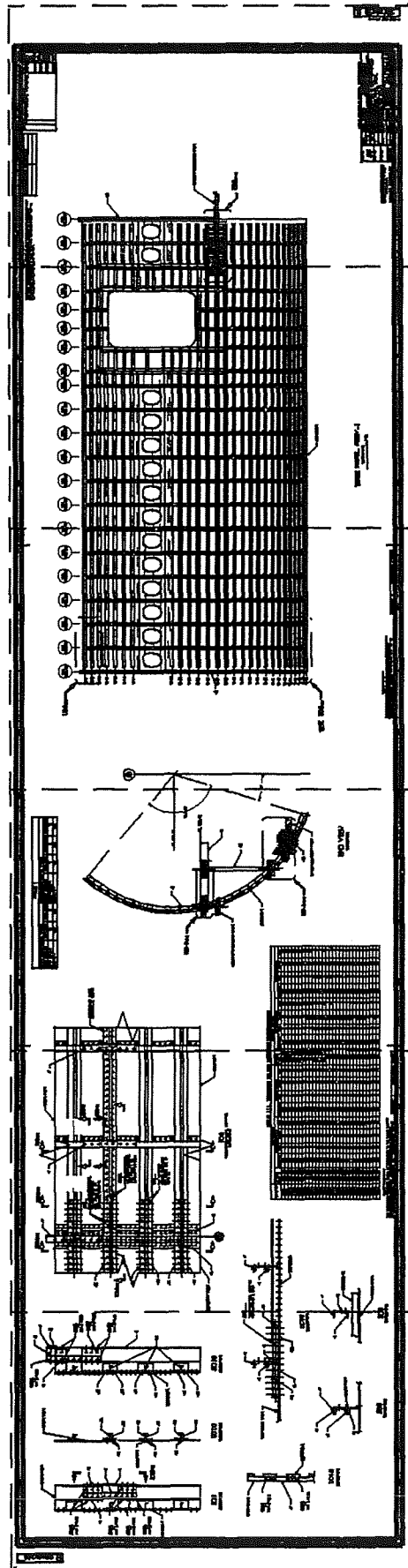
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261X041A 31-

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DATE 11-19-2001 BY 60322
AUTHORITY FROM THE BOM



DETAIL-6
SCALE: 1/1



B-68

B-74

B-73

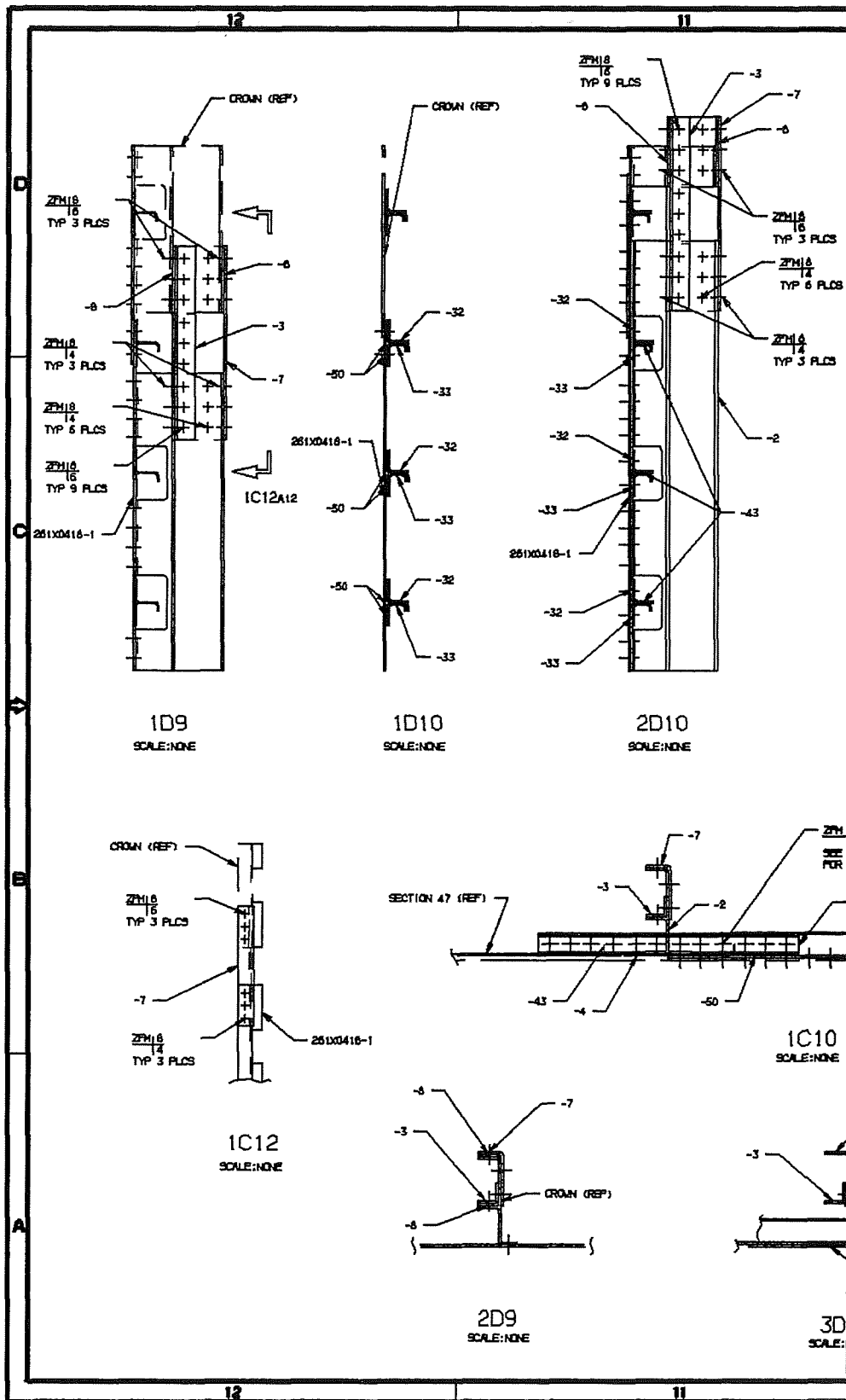
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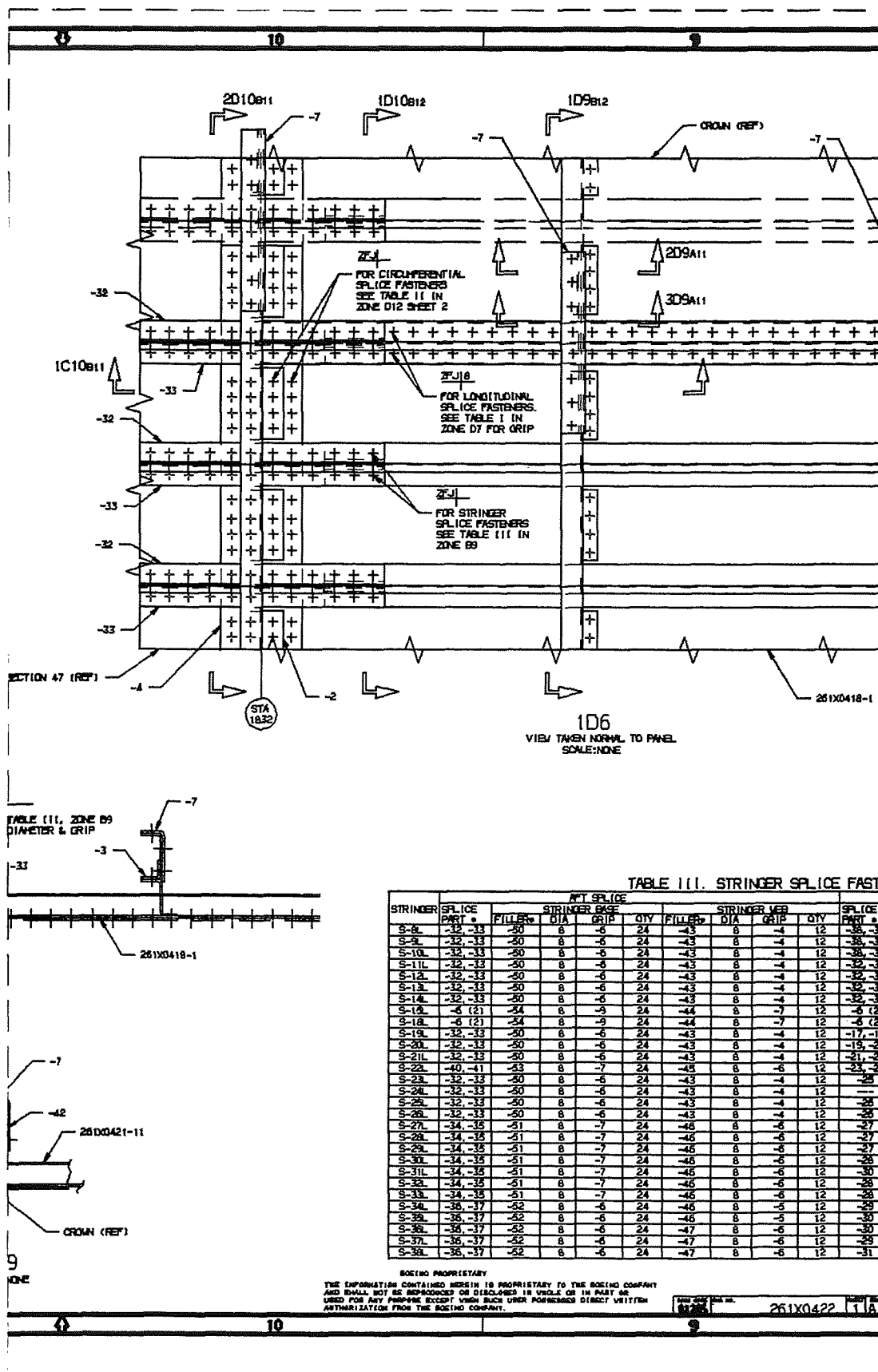
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B-70

B-69

SEIX0455

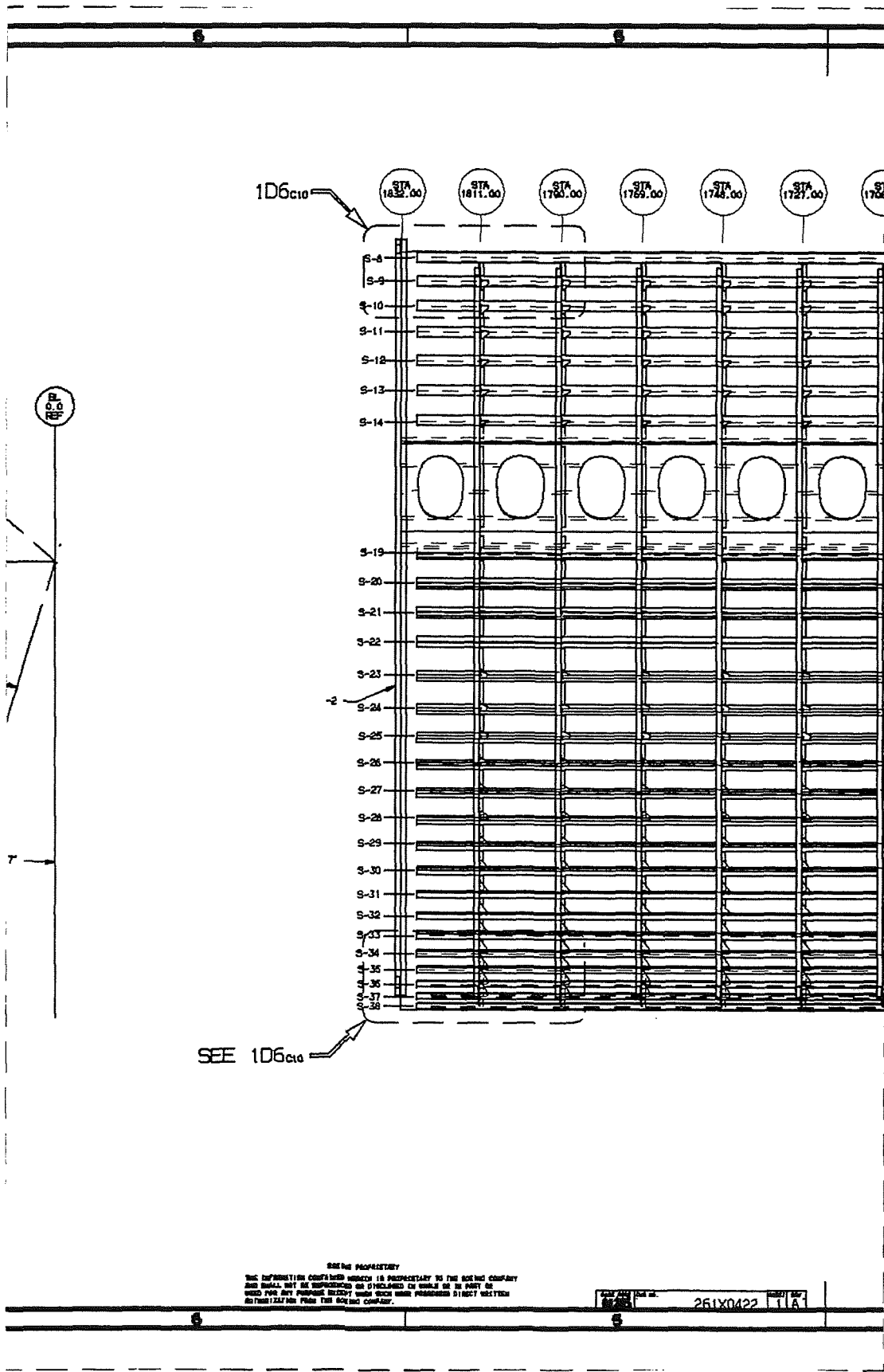




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25100421-12 (S-38)

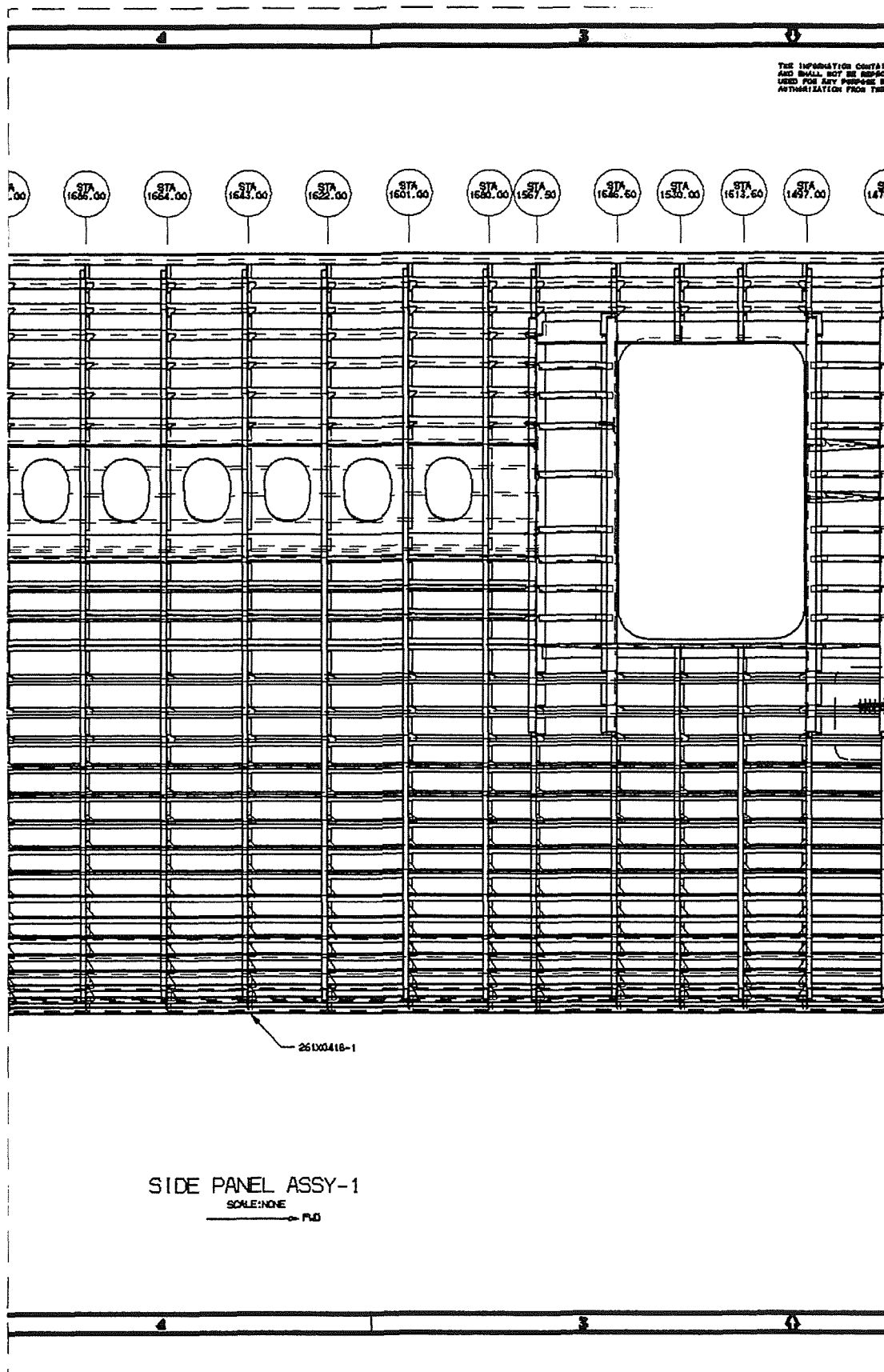


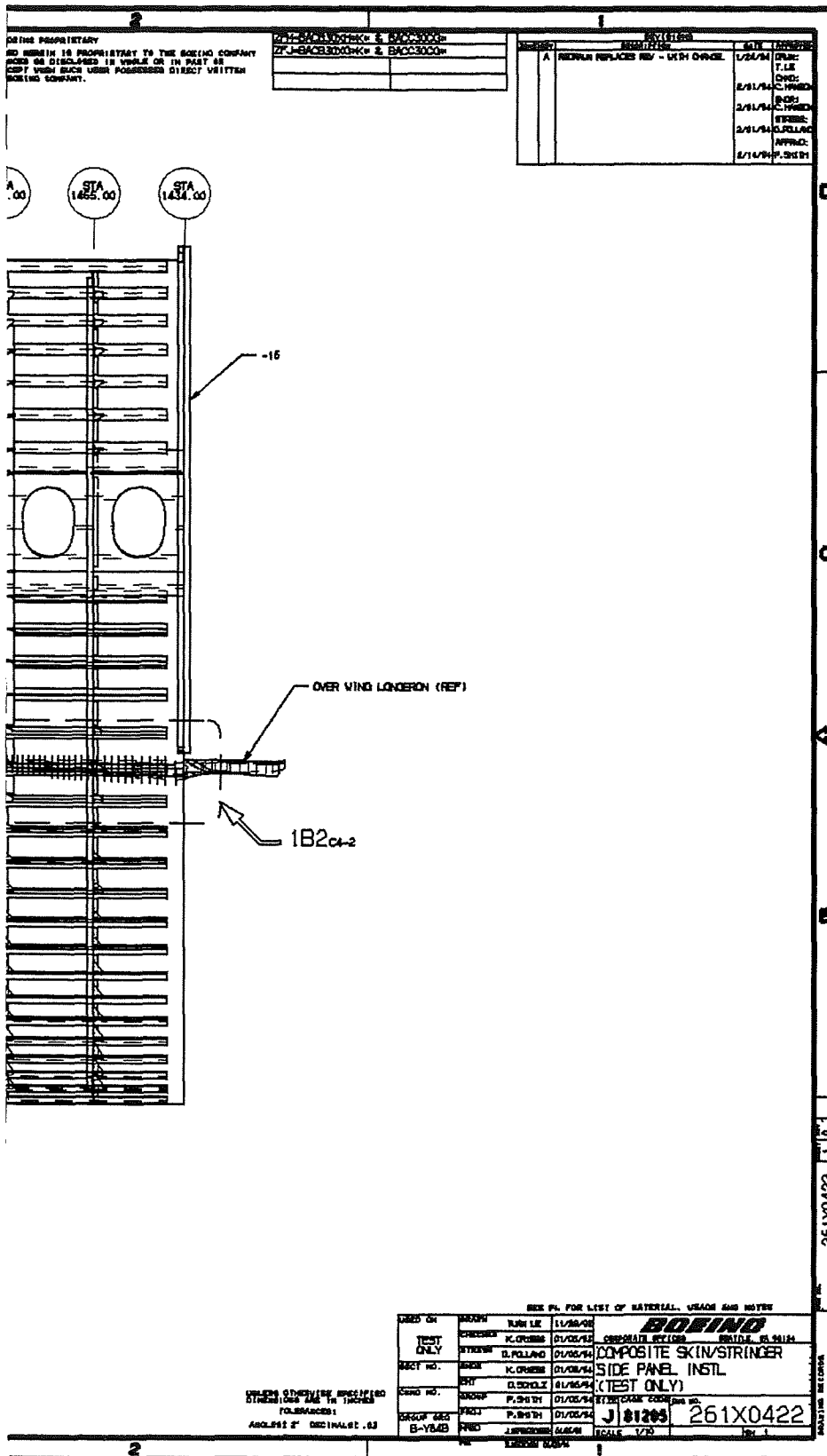
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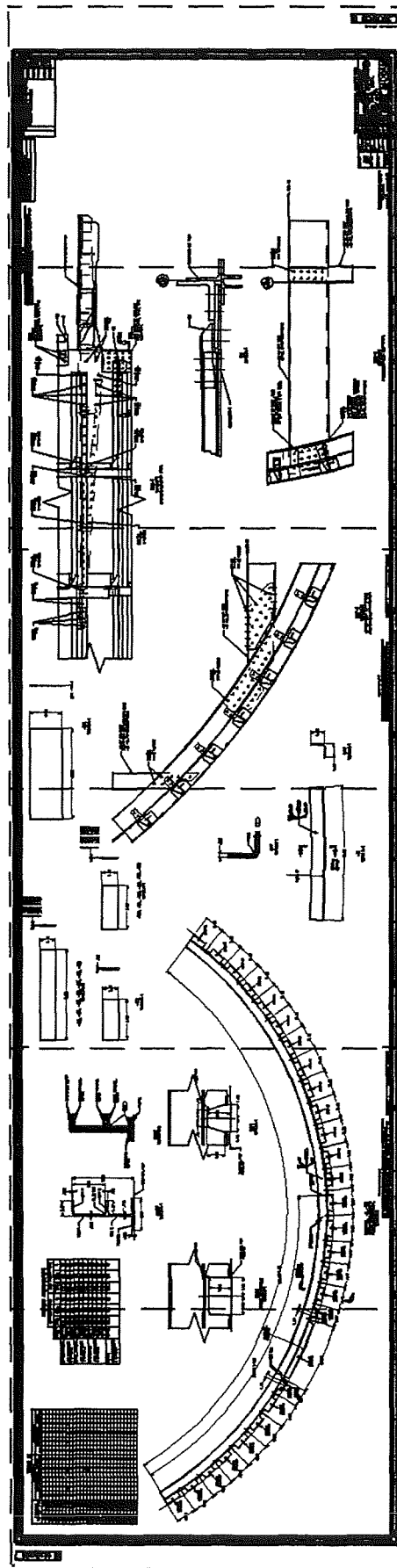
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B-81

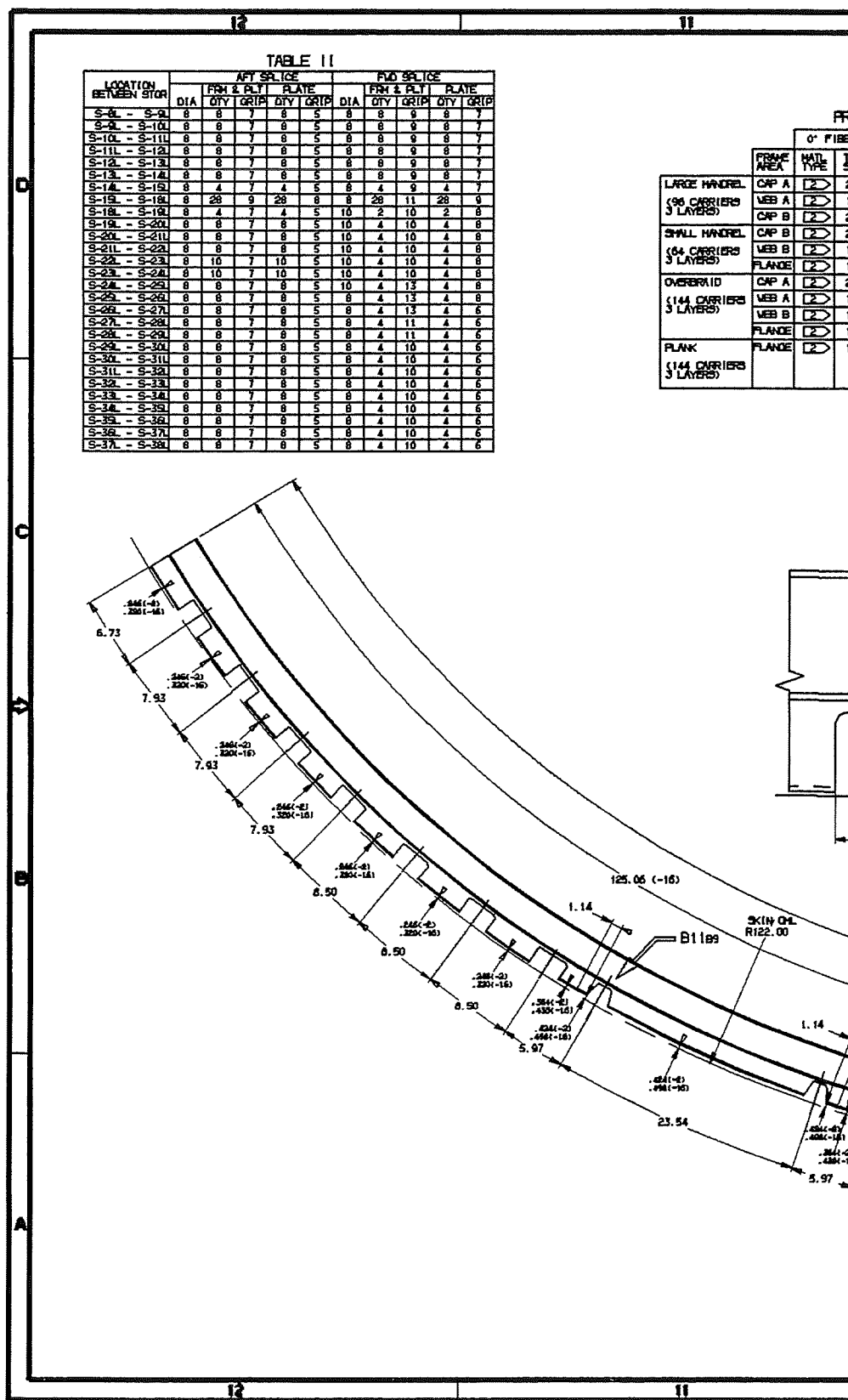
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B-79

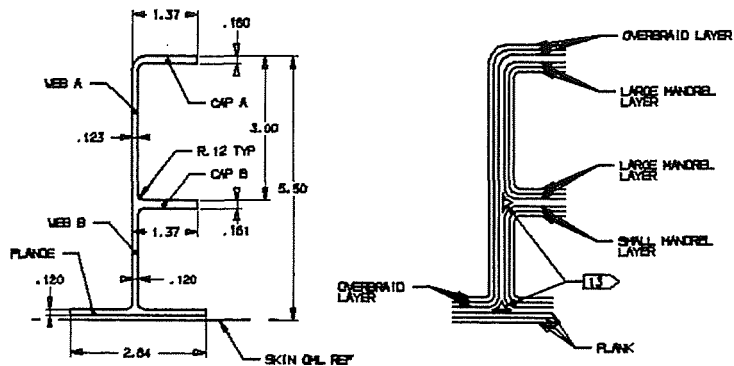
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B-77

B-76

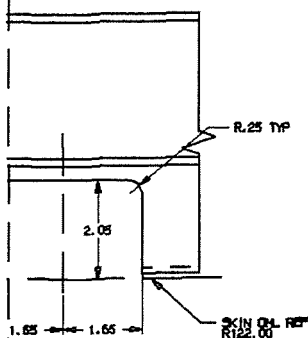


ANGLE		±50° FIBER ANGLE		
IN	%	MAT TYPE	TOW SIZE	%
K	50	2	6K	50
K	33	2	6K	67
K	50	2	6K	50
K	50	2	6K	50
K	33	2	6K	67
K	33	2	6K	67
K	50	2	6K	50
K	33	2	6K	67
K	33	2	6K	67
K	33	2	6K	67

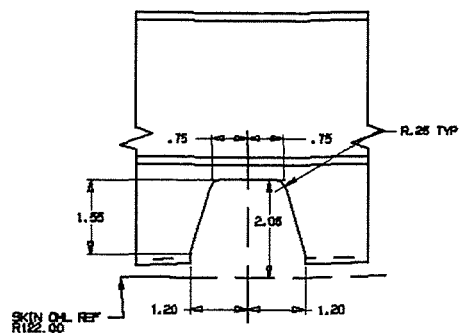


2810
SCALE: 1/1

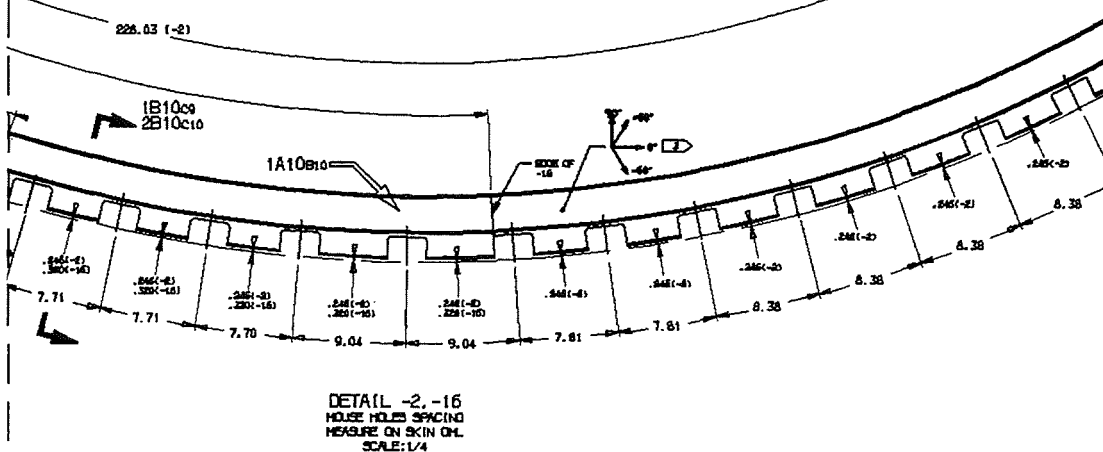
1B10
SCALE:NONE



1A10
TYPICAL HOUSE HOLE
SCALE: 1/1



B11
SCALE:1/1



DETAIL -2, -16
HOUSE HOLES SPACING
MEASURE ON SKIN ONL.
SCALE: 1/4

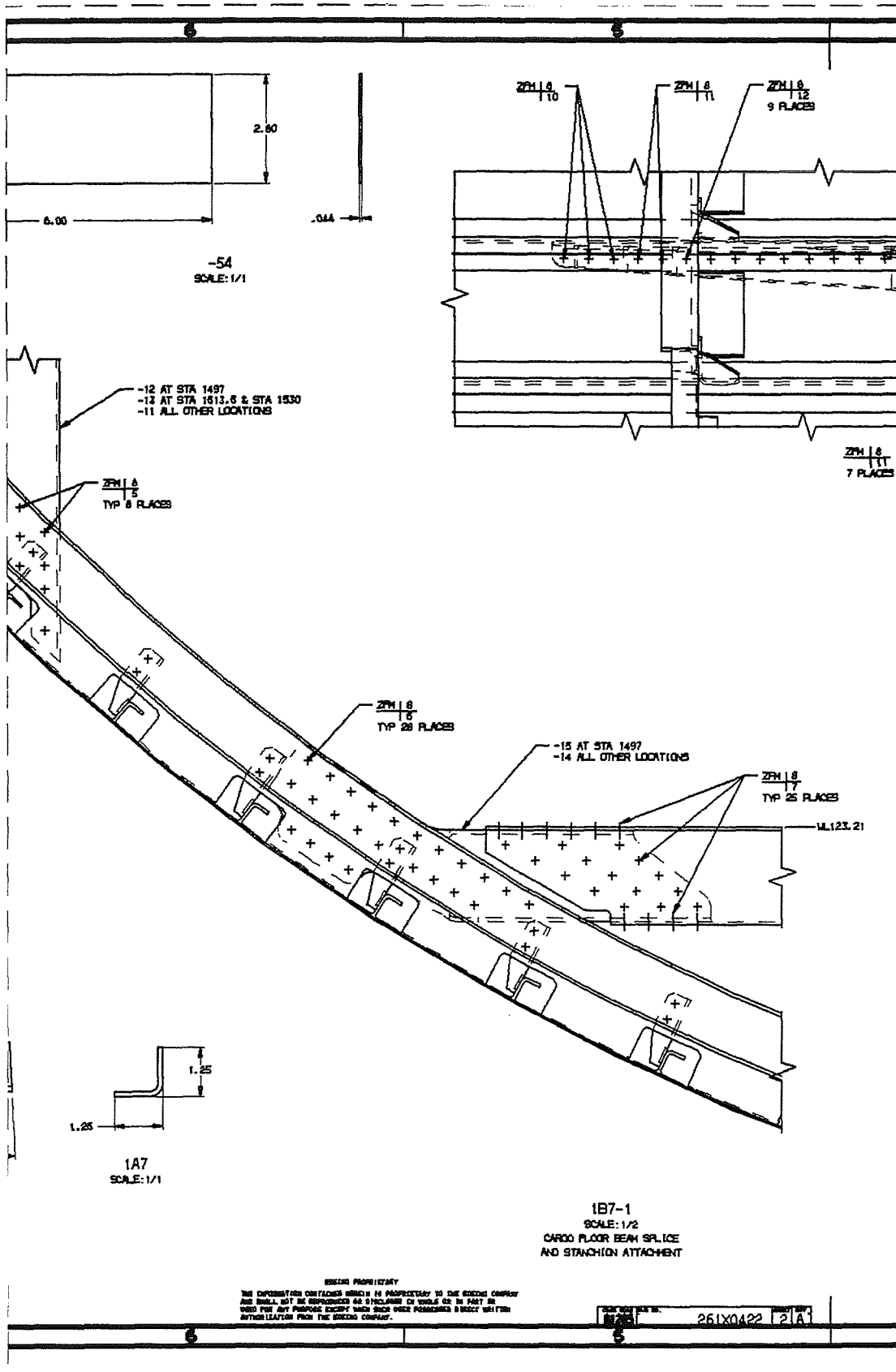
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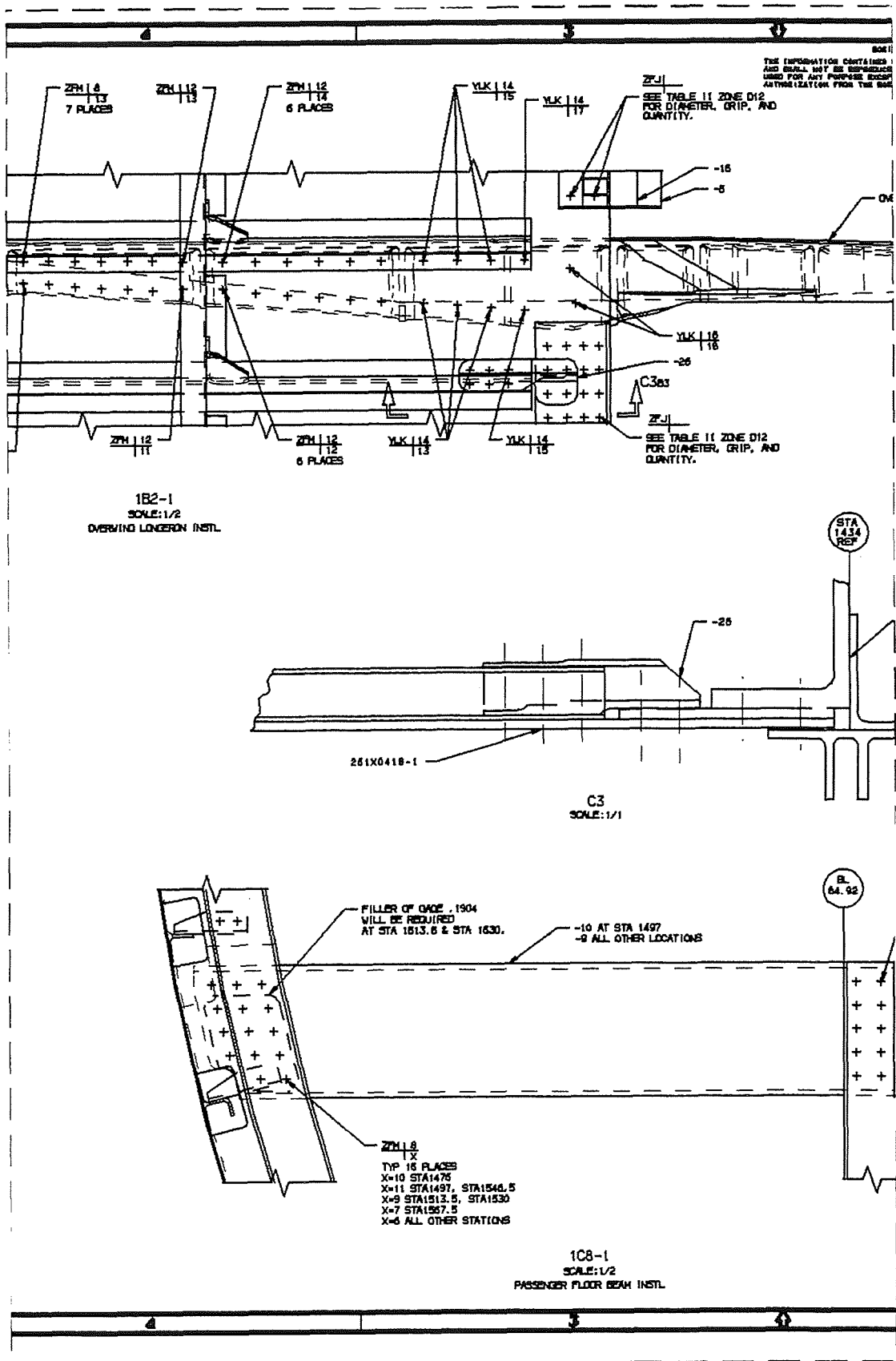
BOOKING PROPRIETARY

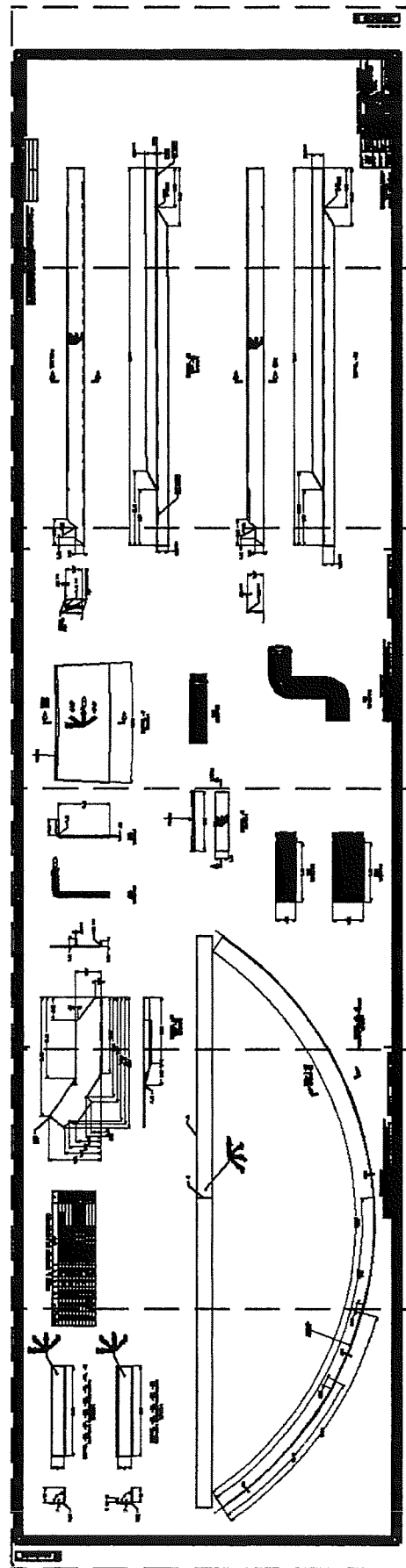
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B-88

B-87

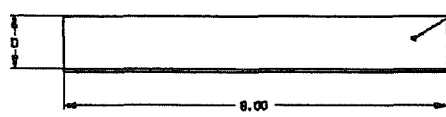
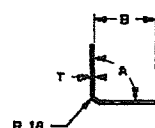
B-86

B-85

B-84

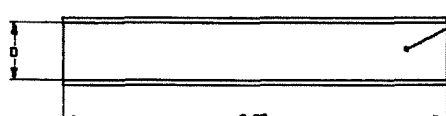
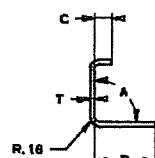
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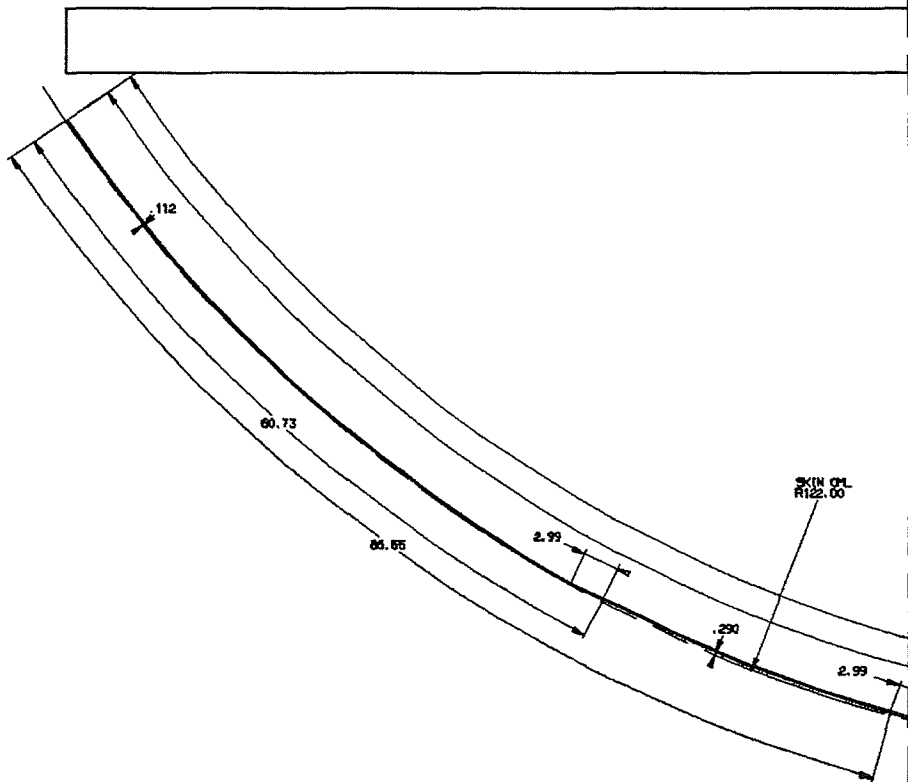


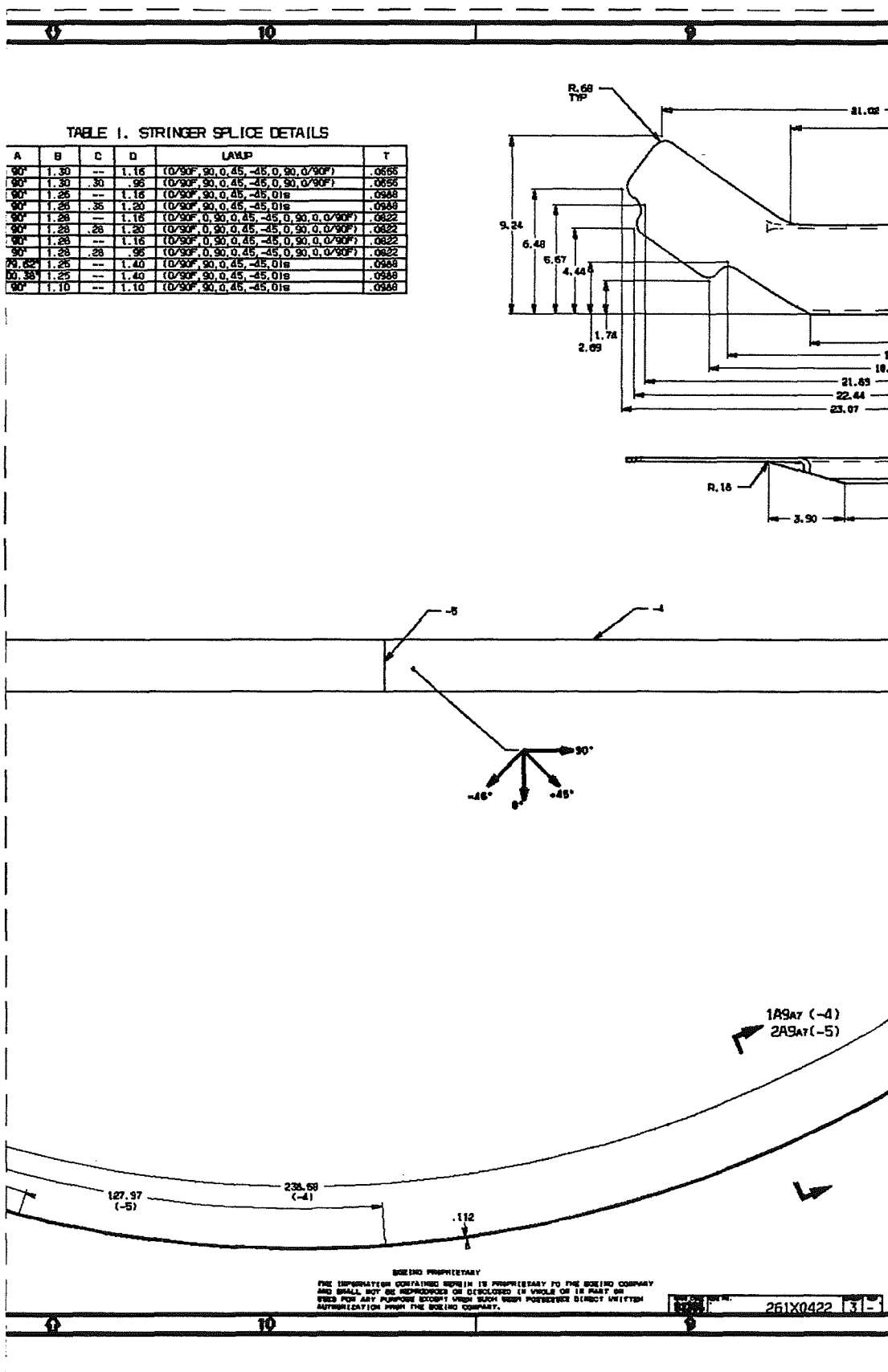
DETAIL -32, -34, -36, -38, -40, -41, -6
SEE TABLE I (ZONE D10) FOR DIMENSIONS
SCALE: 1/1

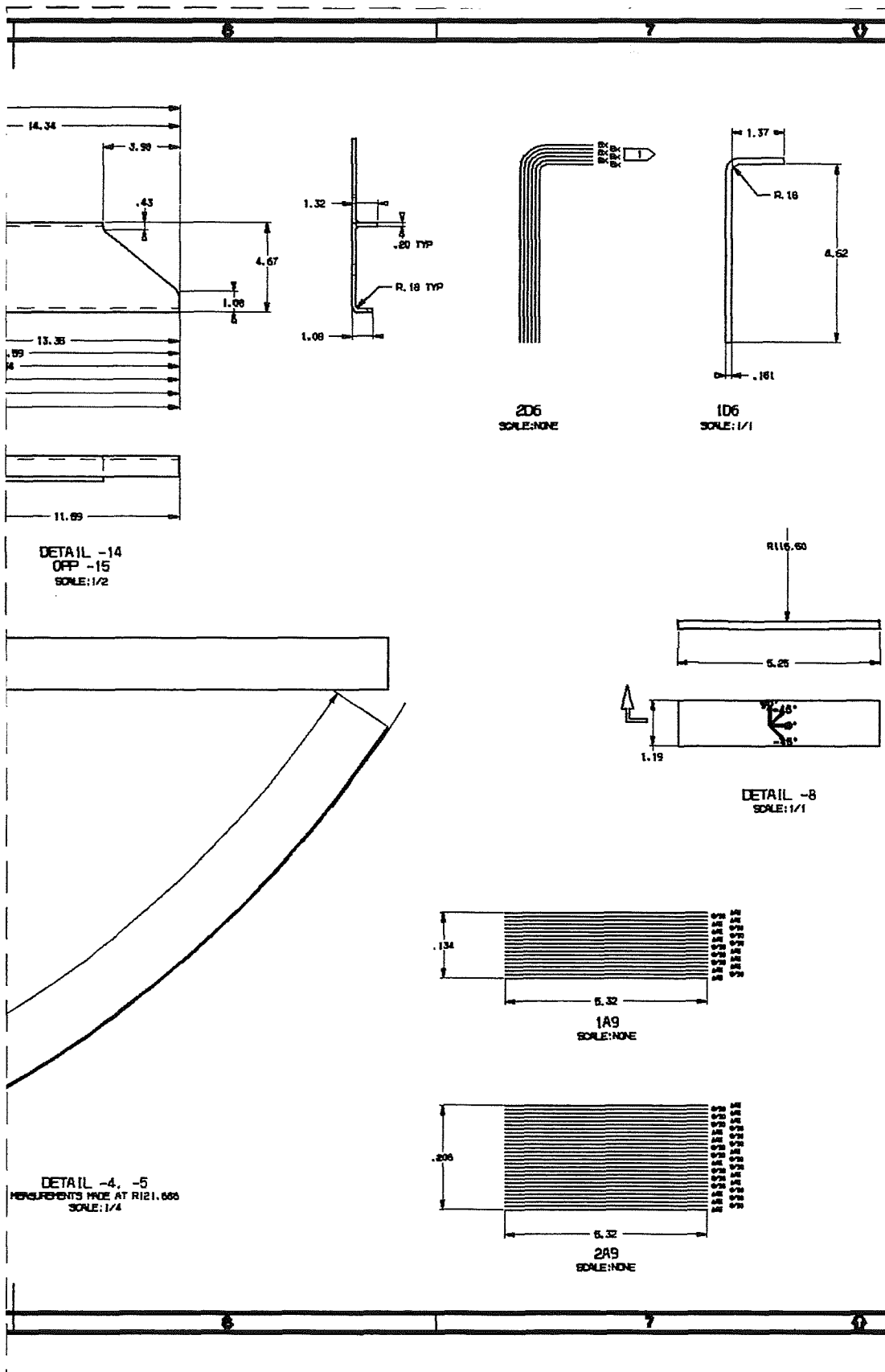
DETAIL
-32
-34
-36
-38
-40
-41
-6

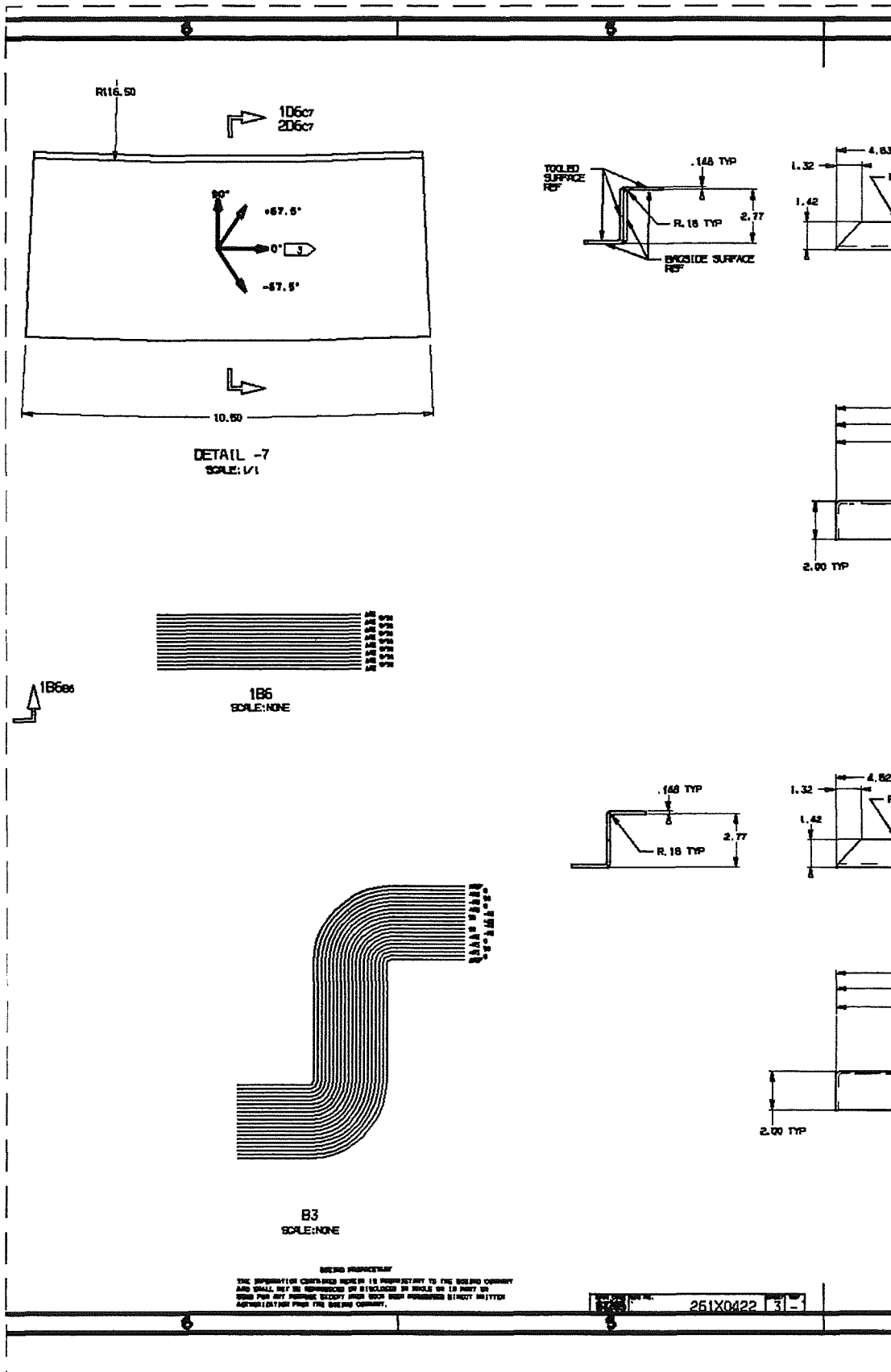


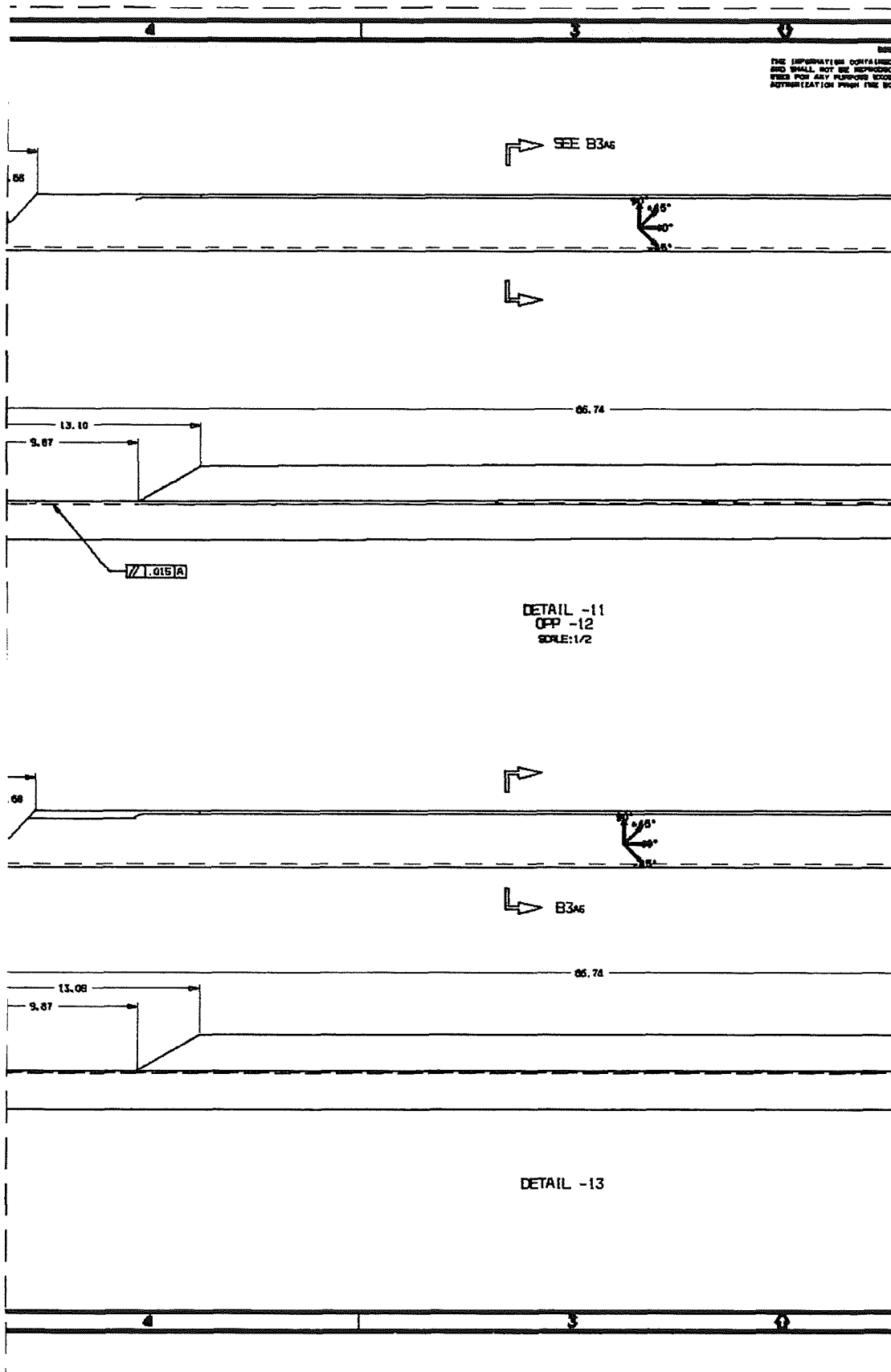
DETAIL -33, -35, -37, -39
SEE TABLE I (ZONE D10) FOR DIMENSIONS
SCALE: 1/1

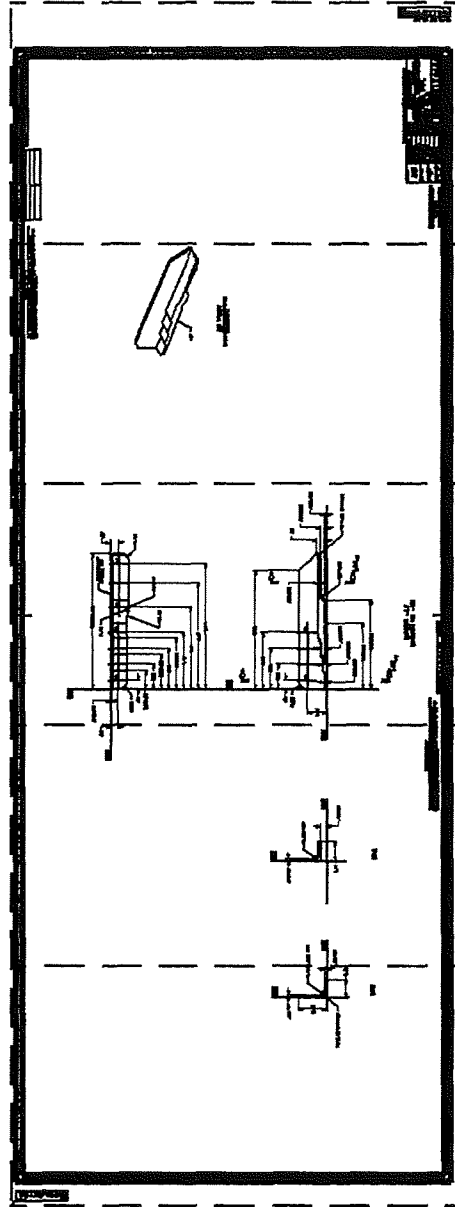












B-89

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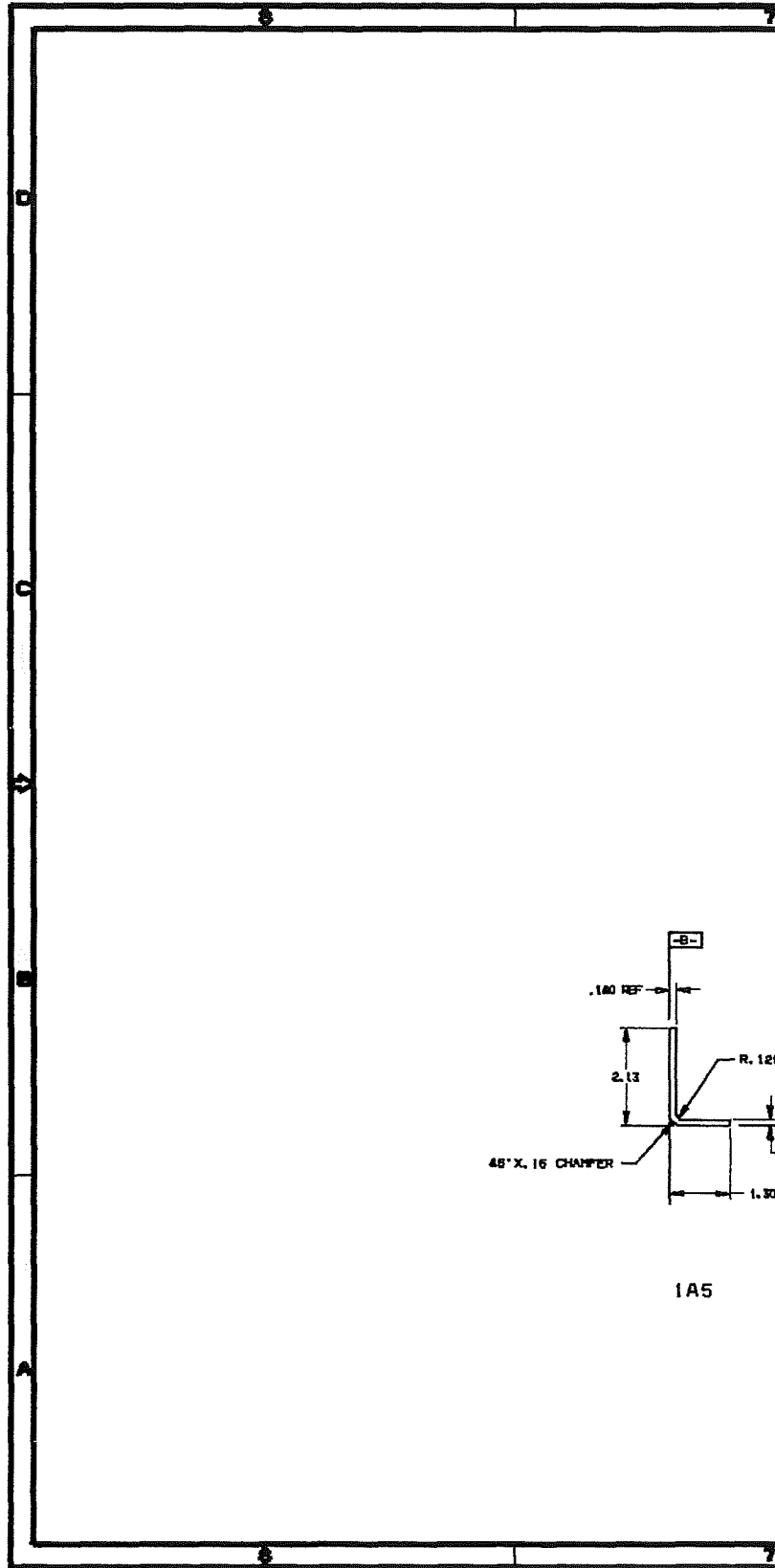
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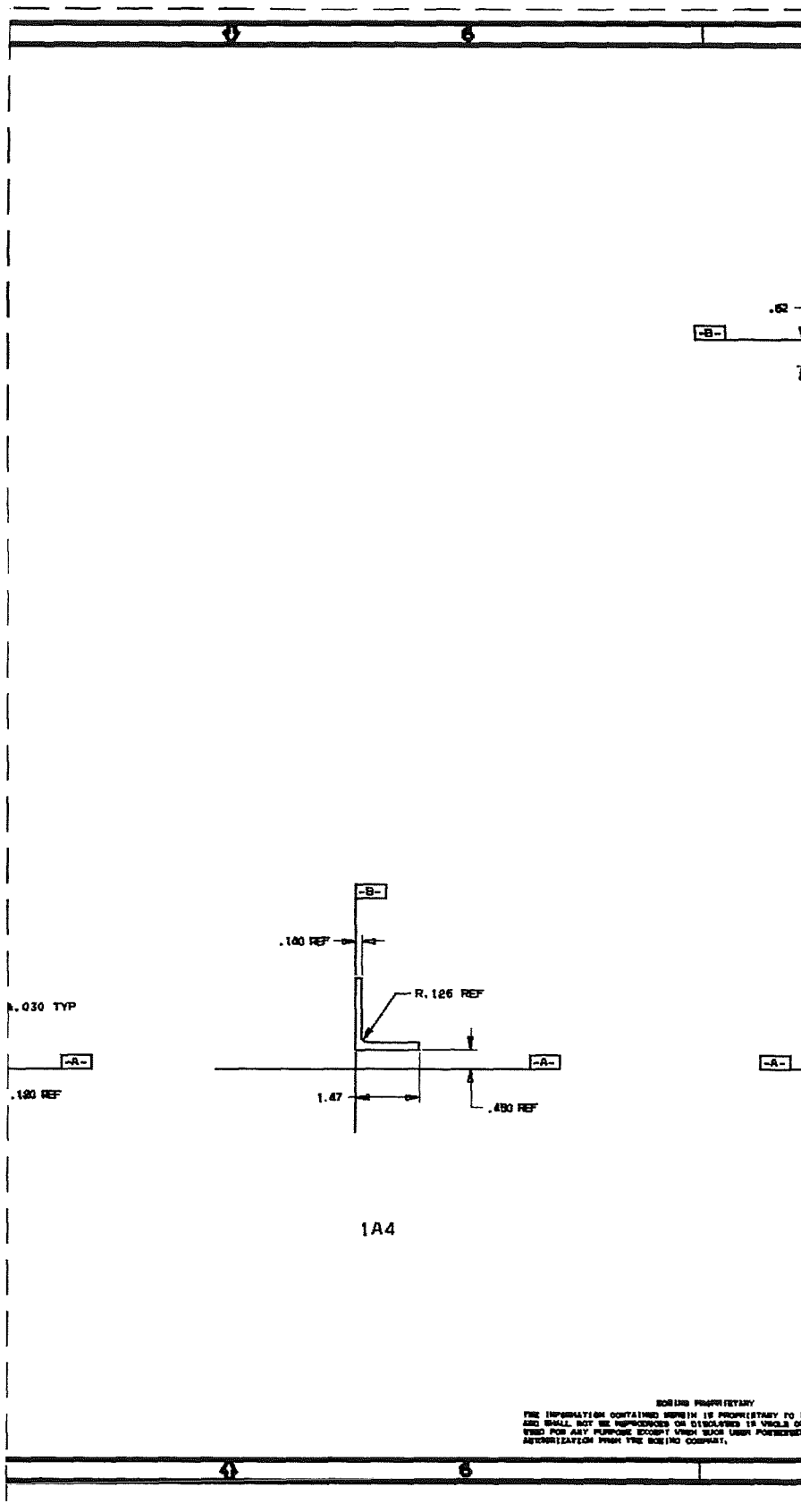
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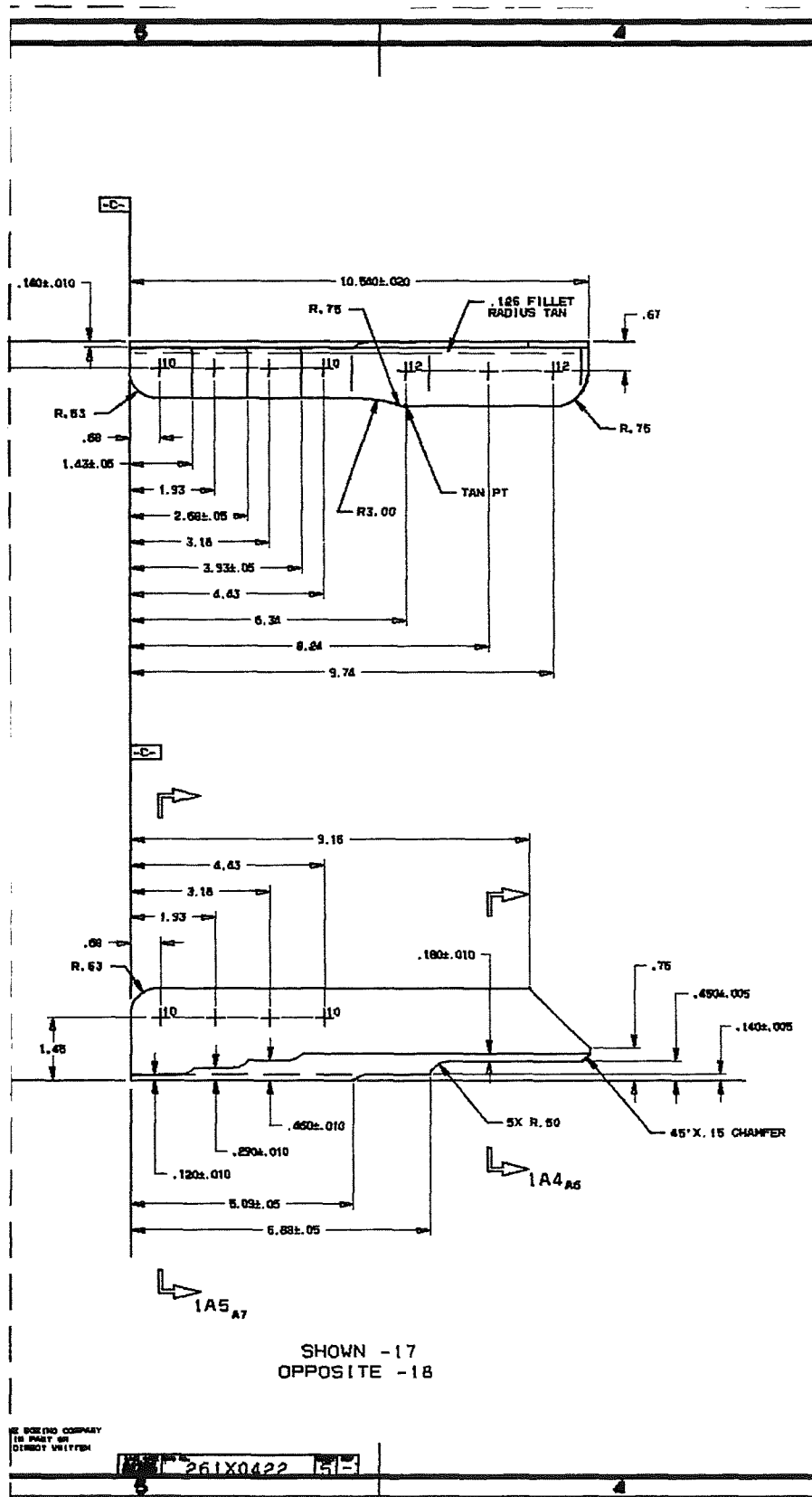
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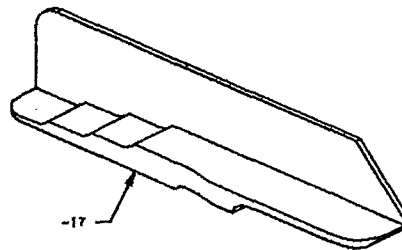
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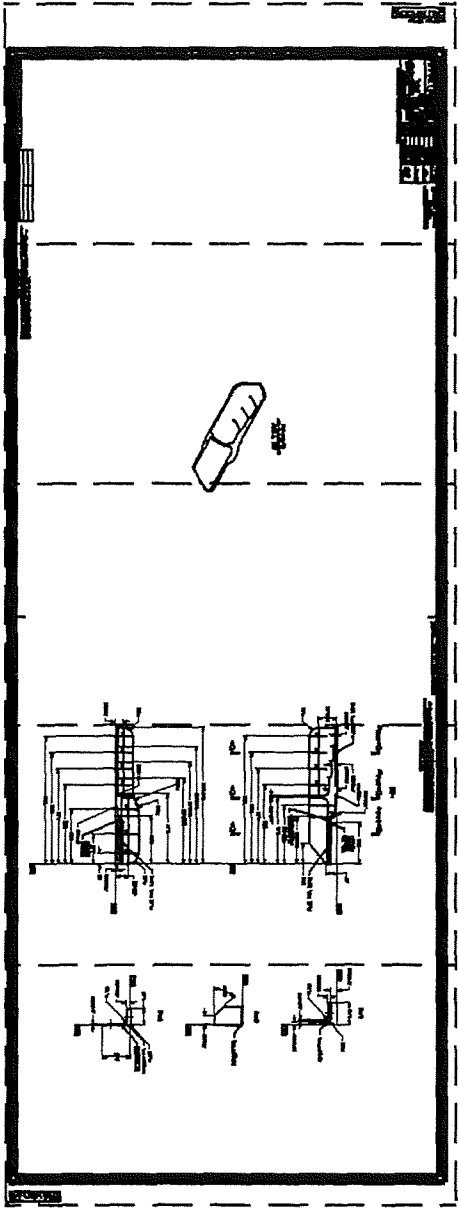




SECRET PROPRIETARY
THE INFORMATION CONTAINED HEREIN IS PROPRIETARY TO THE BUREAU OF AERONAUTICS AND SHALL NOT BE REPRODUCED OR DISCLOSED IN WHOLE OR IN PART FOR ANY PURPOSE EXCEPT UNDER WRITTEN AUTHORIZATION FROM THE BUREAU OF AERONAUTICS.



3D VIEW
ENGINEERING REFERENCE
SCALE: NONE



B-100

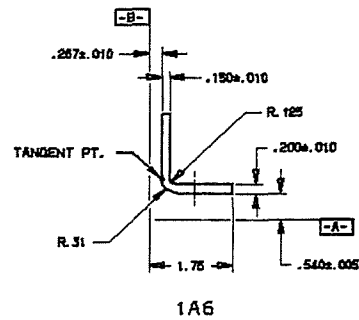
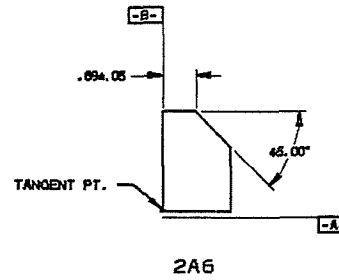
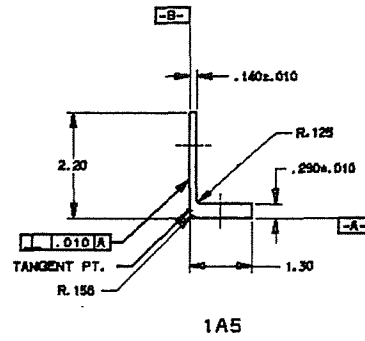
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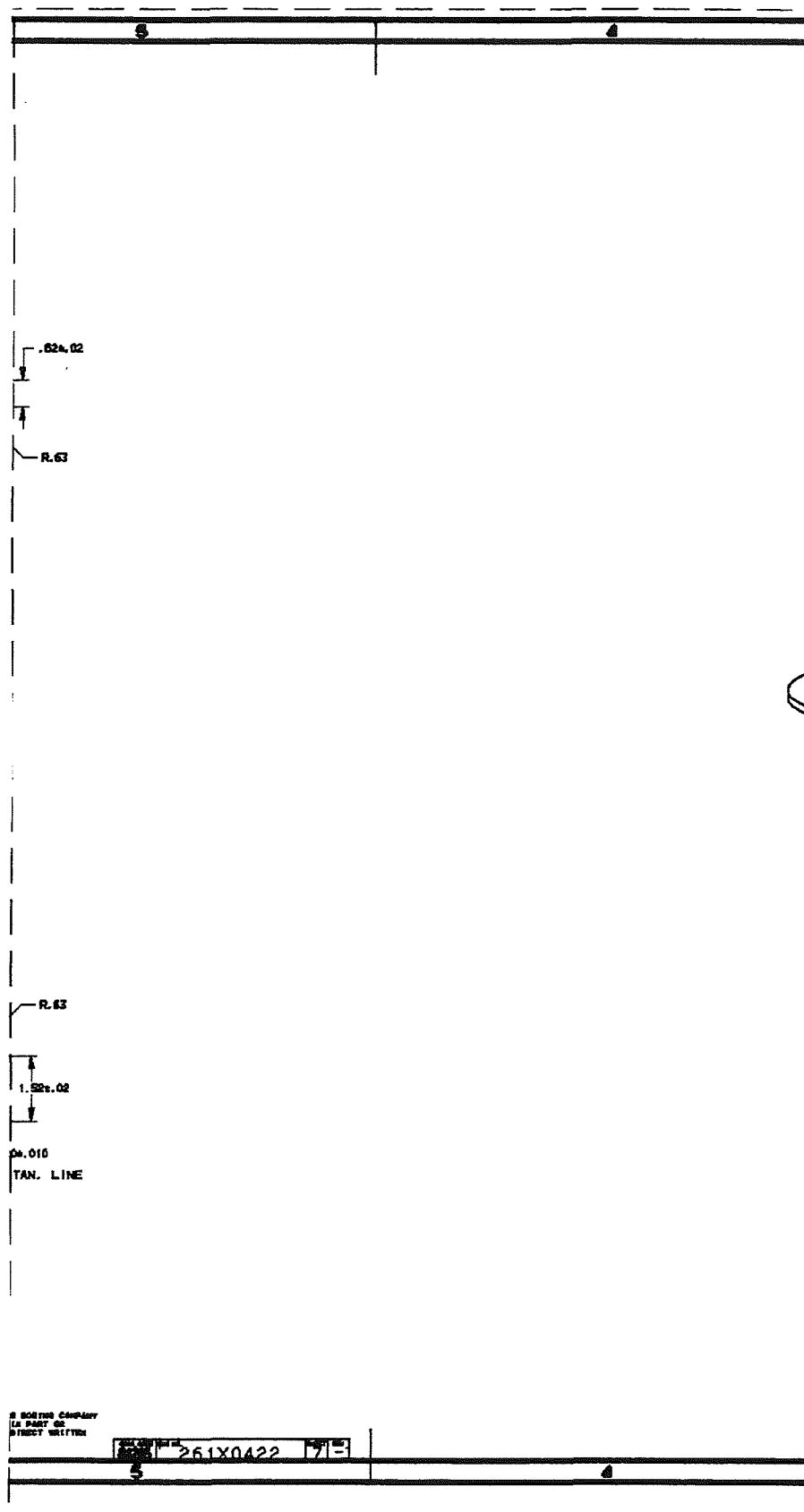
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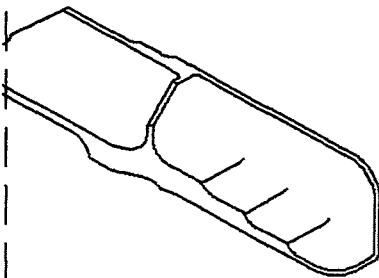
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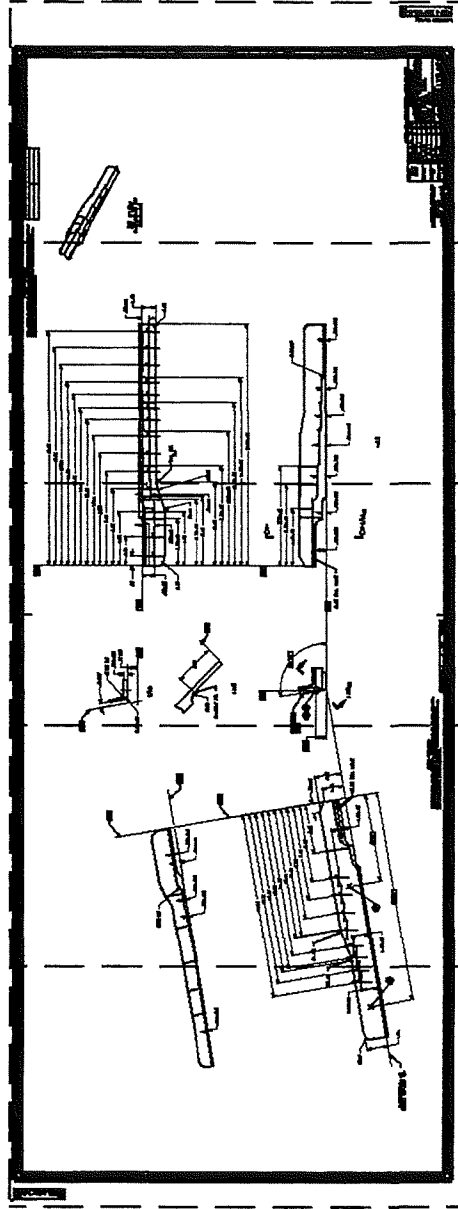
ROTHS PROPRIETARY

THE INFORMATION CONTAINED HEREIN IS PROPRIETARY TO THE ROYAL
AND SHALL NOT BE REPRODUCED OR DISCLOSED IN ANY MANNER
USED FOR ANY PURPOSE EXCEPT WITH ROYAL'S WRITTEN PERMISSION OR
AUTHORIZATION FROM THE ROYAL COMPANY.



3D VIEW
ENGINEERING REF
SCALE: NONE

75540X1a5



B-101

B-106

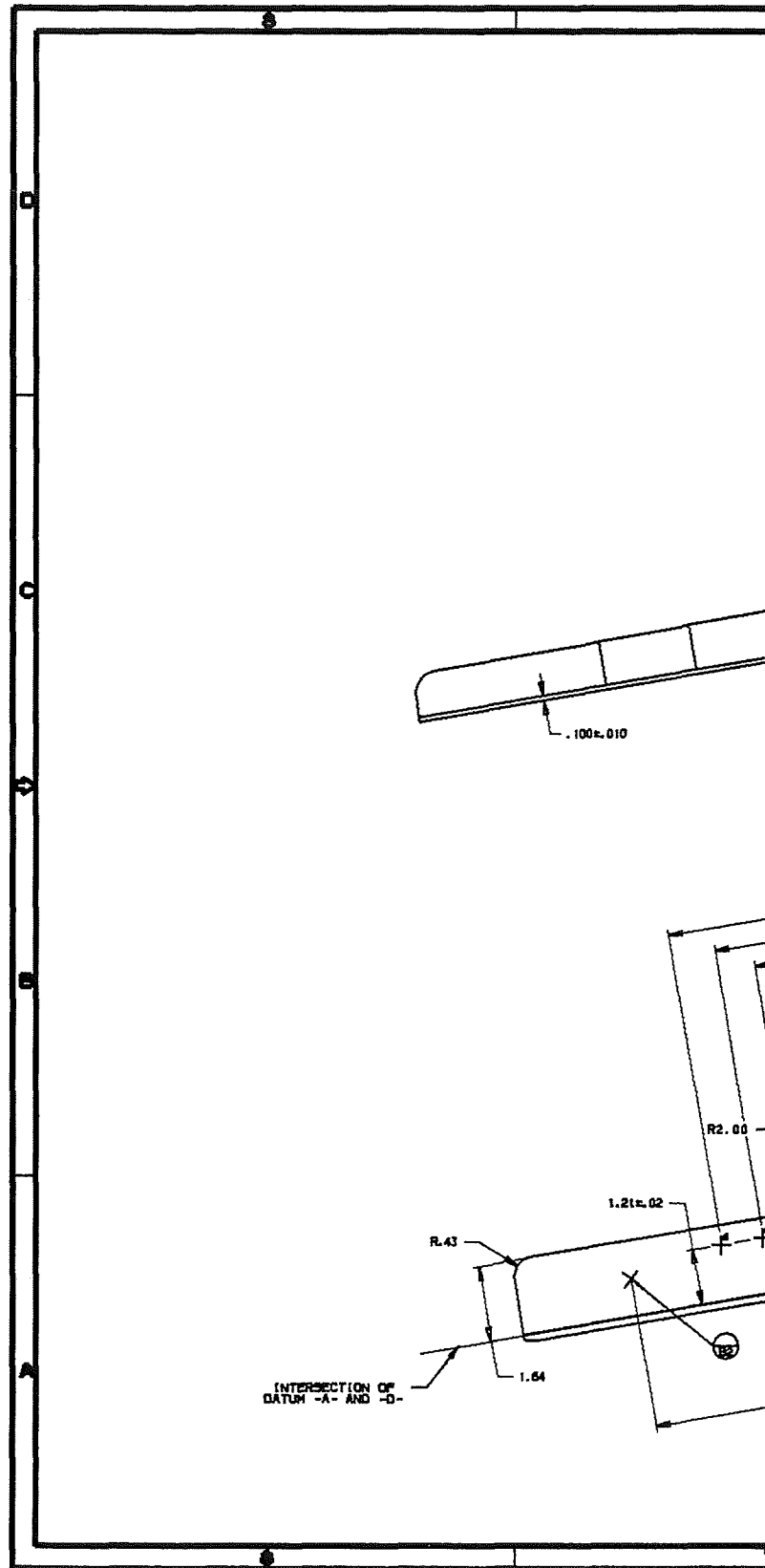
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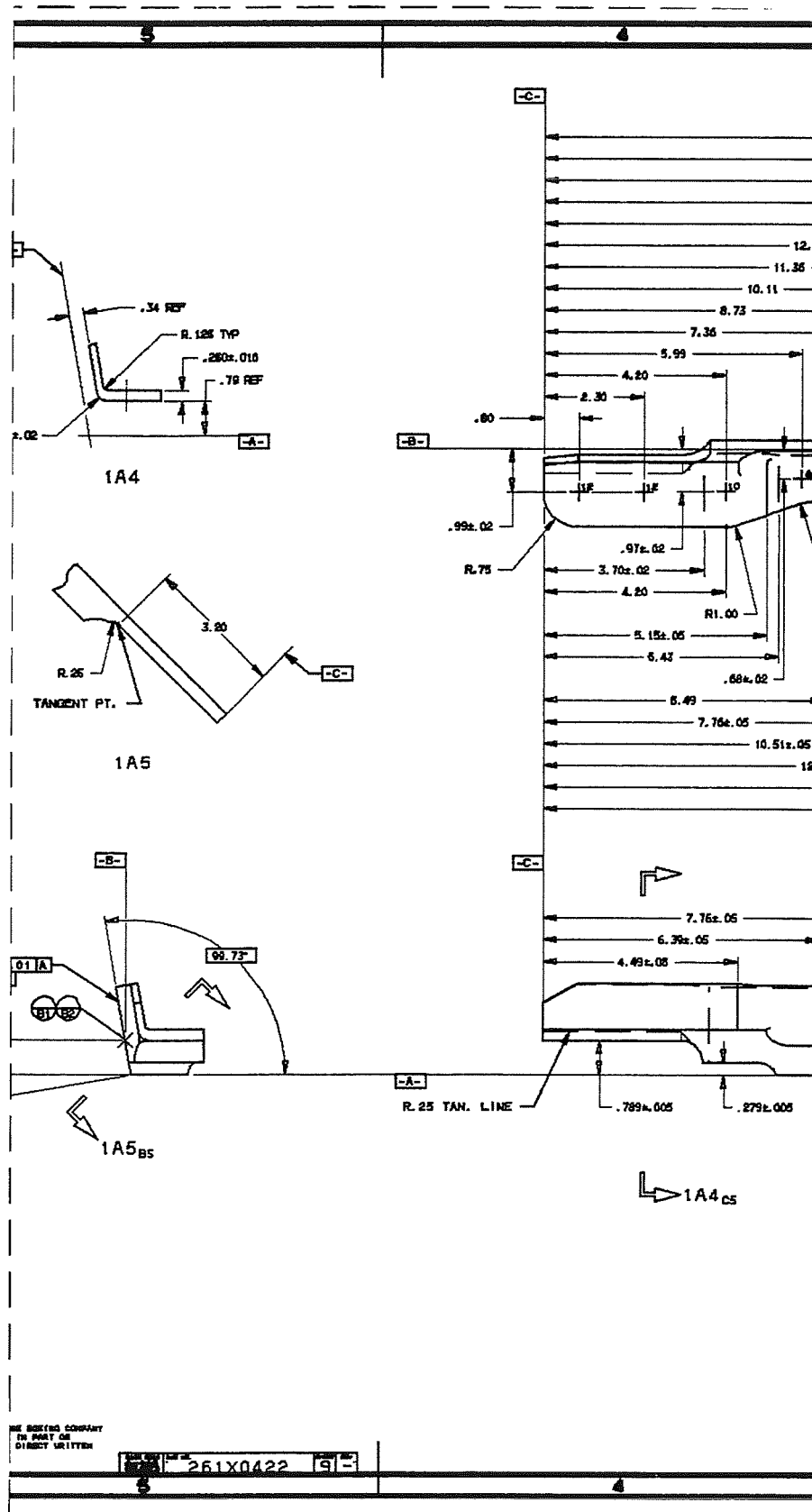
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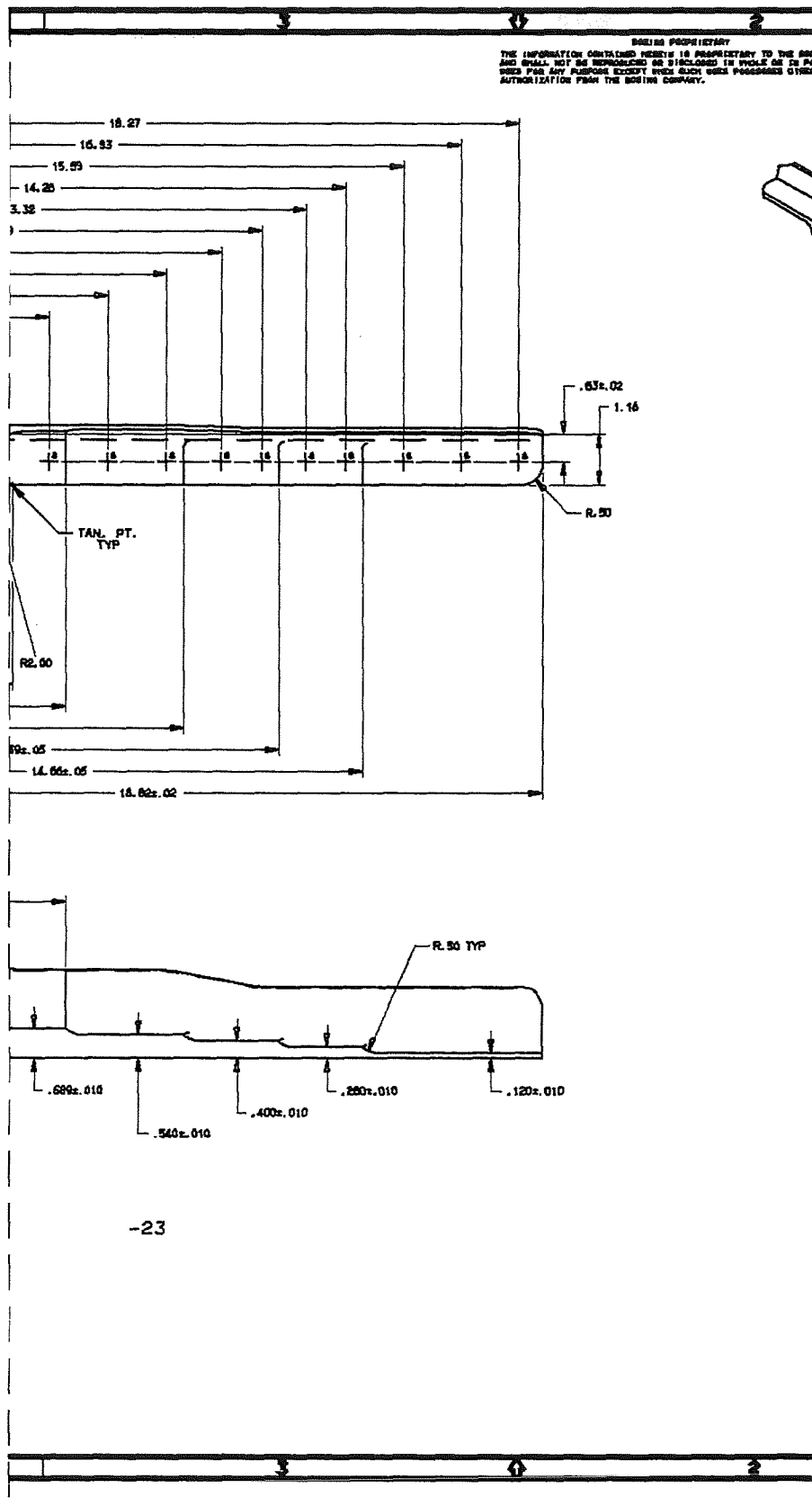
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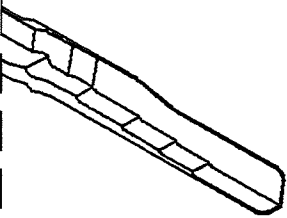
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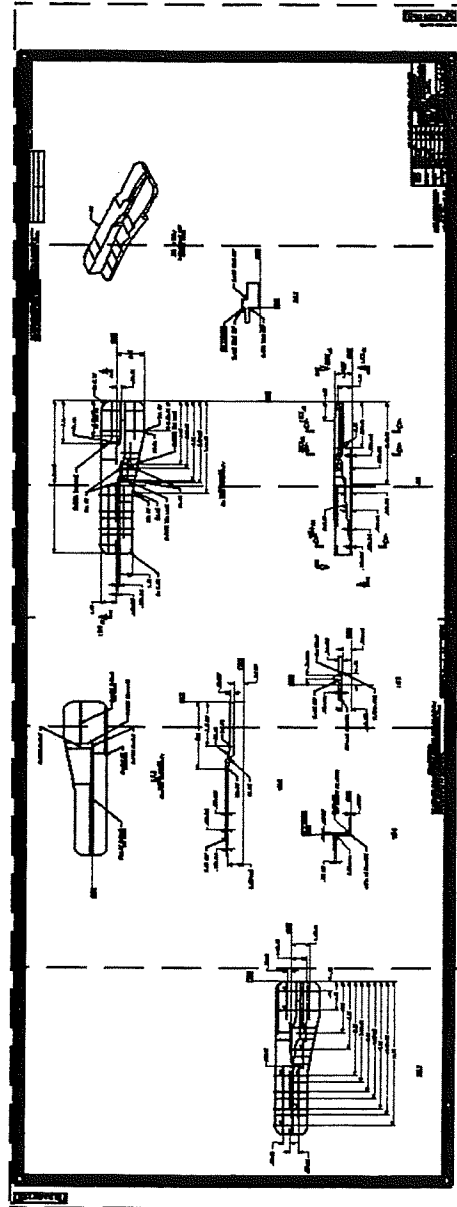




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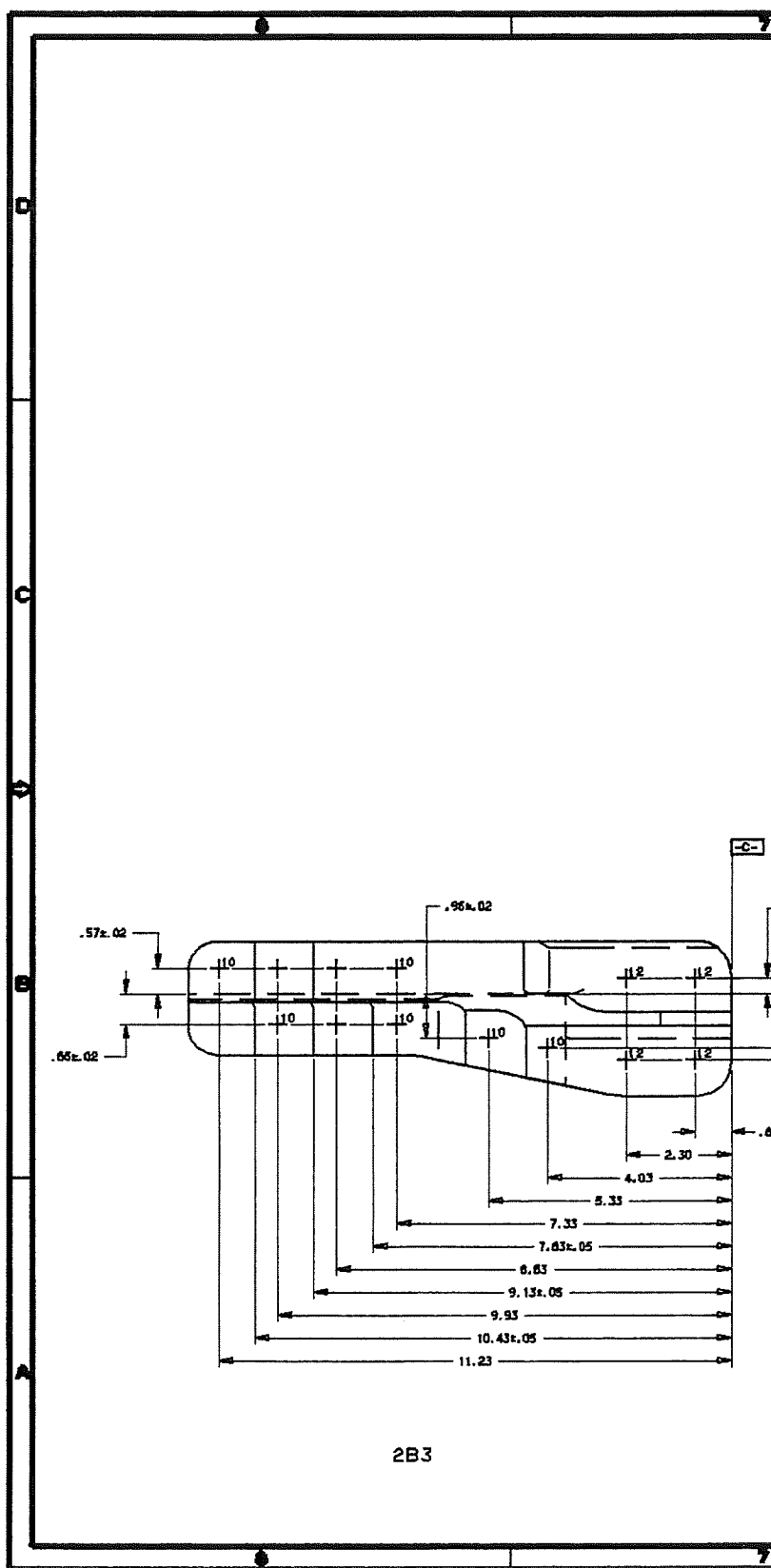
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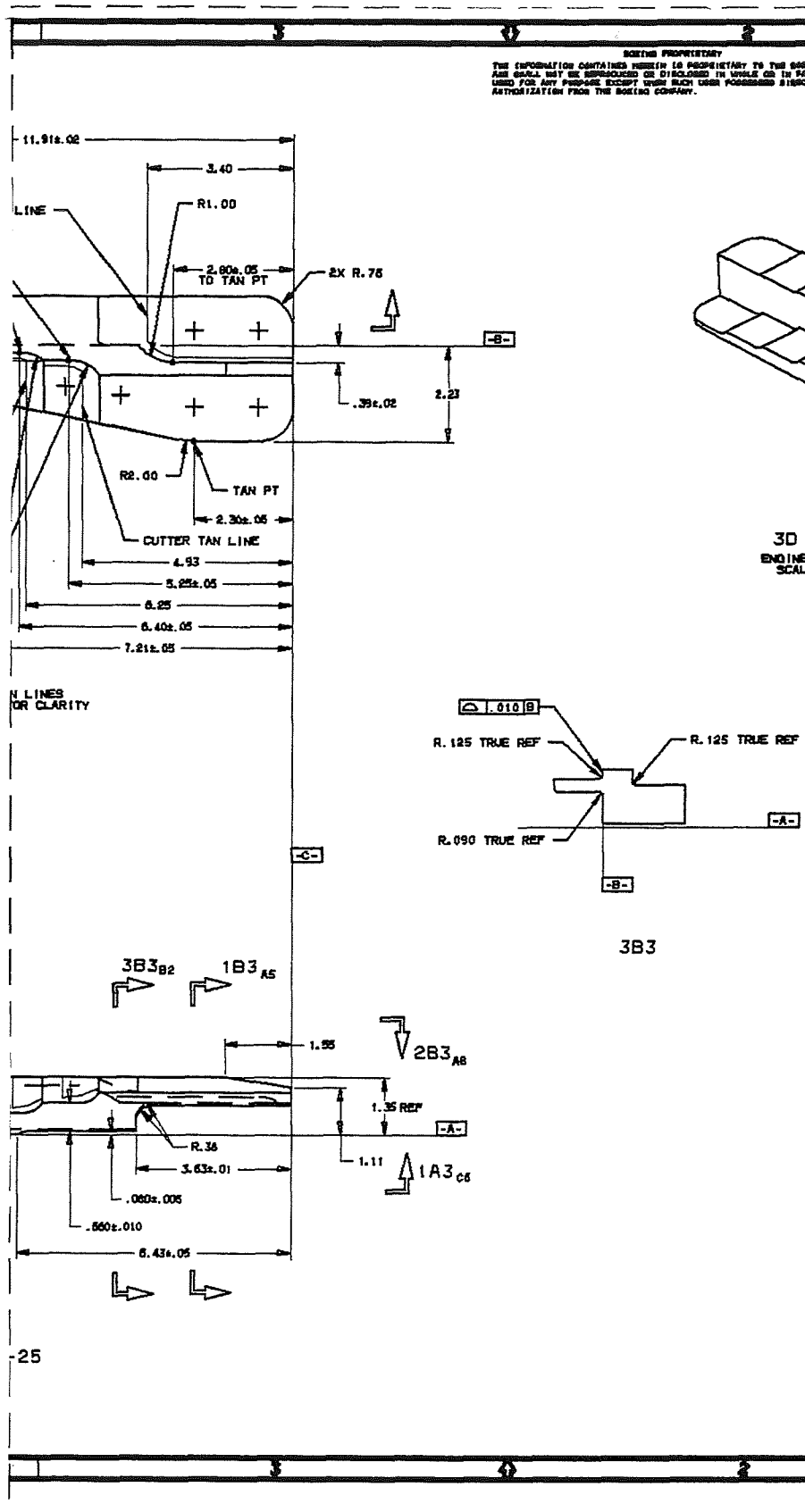
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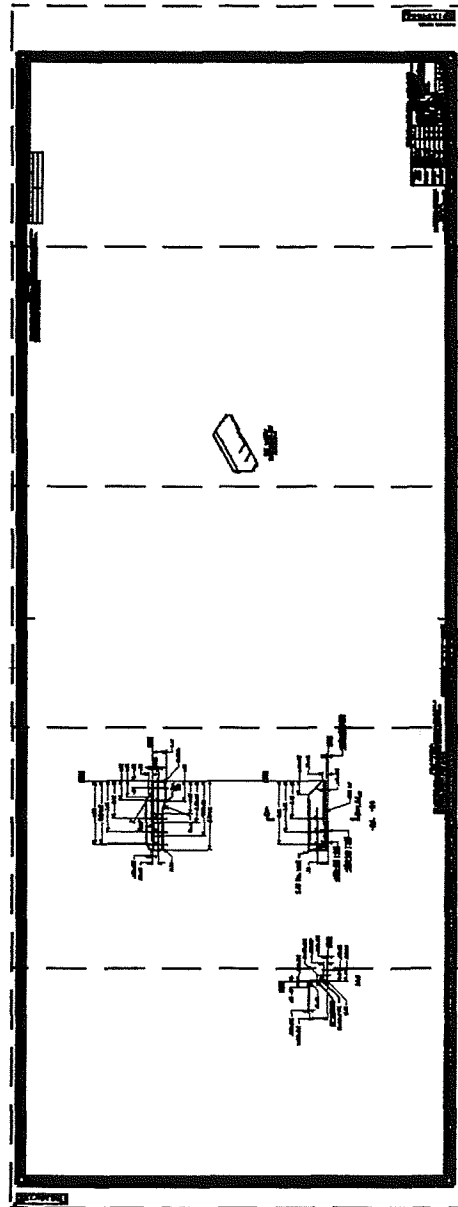
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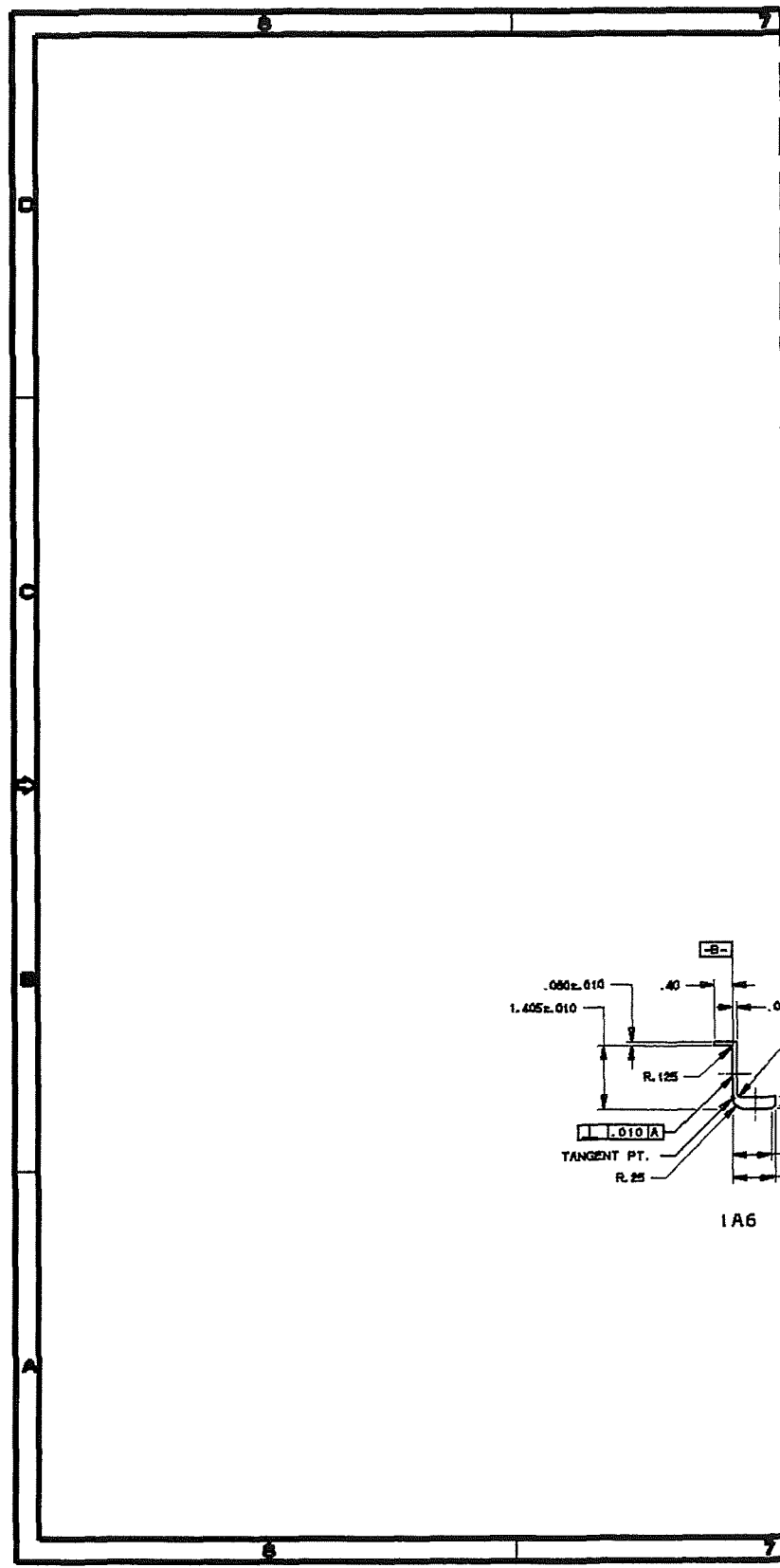
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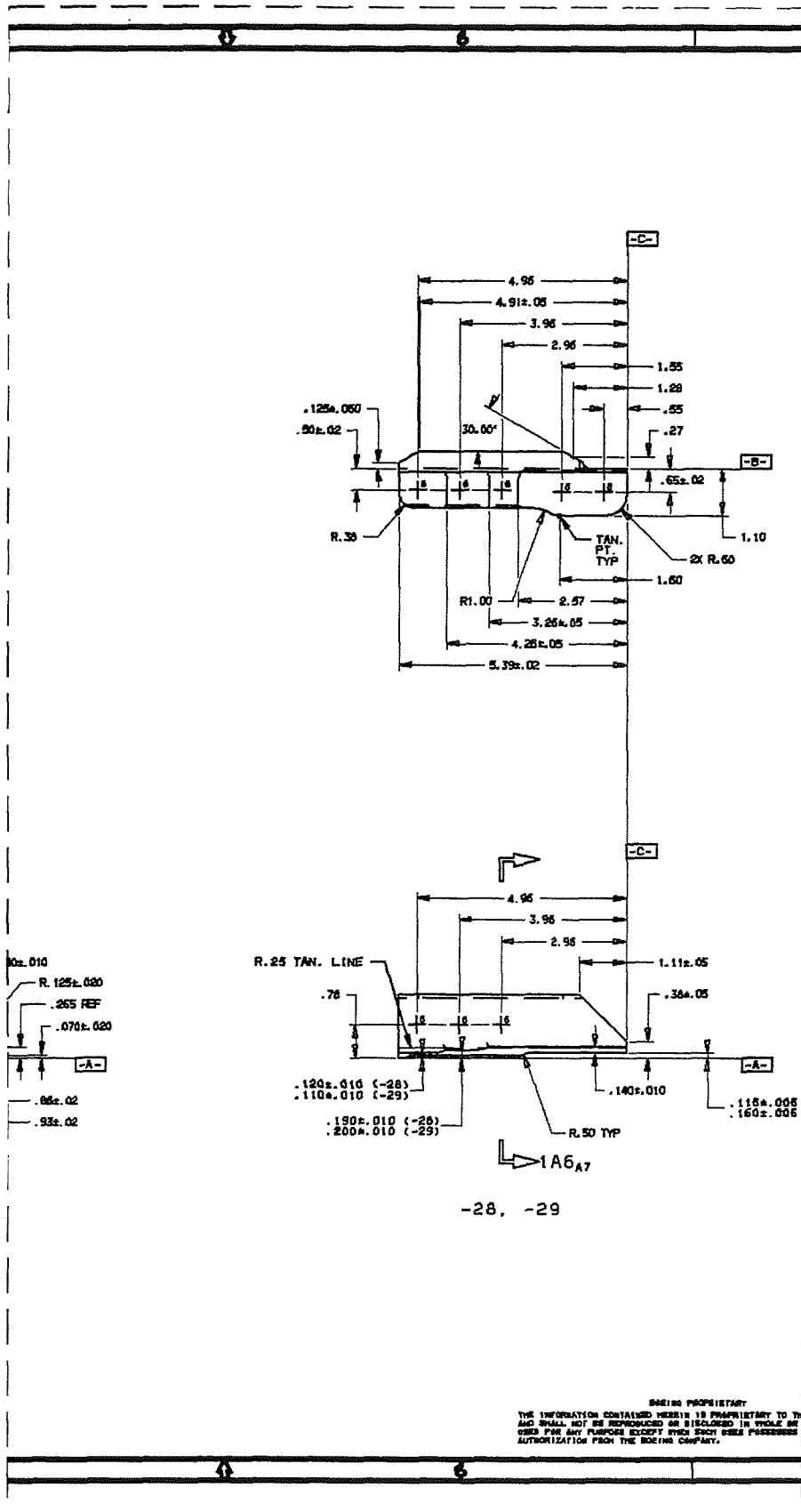
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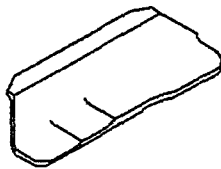
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BOEING PROPRIETARY
THE INFORMATION CONTAINED HEREIN IS PROPRIETARY TO THE BOEING COMPANY AND SHALL NOT BE REPRODUCED OR DISCLOSED IN ANY MANNER OR BY ANY MEANS FOR ANY PURPOSES WITHOUT EXPRESS WRITTEN AUTHORIZATION FROM THE BOEING COMPANY.



3D VIEW
ENGINEERING REF
SCALE: NONE

APPENDIX C

COST AND WEIGHT DATA

SIDE PANEL COST & WEIGHT BREAKDOWN
FAMILY B (SKIN-STRINGER) - GLOBAL EVALUATION BASELINE
(Weight in lbs per shipset, Cost in \$ per 300 shipsets)

C	PART #	DESCRIPTION	QTY	WEIGHT	RECURRING LABOR	RECURRING MATERIAL	NON RECURRING	TOTAL
1	261X0421-2	SKIN	1	780.7	\$1,983,000	\$8,263,100	\$294,113	\$10,540,213
2	261X0421-11	"J" STRINGER	2	12.9	\$468,000	\$261,922	\$249,016	\$978,938
2	261X0421-12	"J" STRINGER	1	4.2	\$165,000	\$85,882	\$135,772	\$386,654
2	261X0421-13	"J" STRINGER	7	3.5	\$167,000	\$70,560	\$135,772	\$373,332
2	261X0421-14	"J" STRINGER	4	16.7	\$619,000	\$341,482	\$475,506	\$1,435,988
2	261X0421-15	"J" STRINGER	2	7.8	\$319,000	\$158,371	\$249,016	\$726,387
2	261X0421-16	"J" STRINGER	2	19.8	\$468,000	\$407,722	\$249,016	\$1,124,738
2	261X0421-17	"J" STRINGER	2	12.2	\$468,000	\$251,222	\$249,016	\$968,238
2	261X0421-18	"J" STRINGER	7	56.4	\$1,591,800	\$1,015,200	\$815,239	\$3,422,239
2	261X0421-19	"J" STRINGER	4	57.8	\$896,000	\$1,331,712	\$475,506	\$2,703,218
2	261X0421-20	"J" STRINGER	1	6.1	\$162,000	\$145,814	\$135,772	\$443,586
2	261X0421-21	"J" STRINGER	1	1.0	\$22,000	\$25,776	\$135,772	\$183,548
5	AS92CC141-1,2	WINDOW BELT	2	88.0	\$2,428,000	\$1,881,000	\$875,292	\$5,184,292
3	261X0420-1	"F" FRAME	9	119.2	\$4,372,200	\$1,191,523	\$1,045,255	\$6,608,978
3	261X0420-2	"F" FRAME	1	13.2	\$485,700	\$131,947	\$142,145	\$759,792
3	261X0420-3	"F" FRAME	1	13.2	\$485,700	\$131,947	\$142,145	\$759,792
3	261X0420-4	"F" FRAME	1	13.2	\$485,700	\$131,947	\$142,145	\$759,792
3	261X0420-5	"F" FRAME	2	11.7	\$531,600	\$120,393	\$96,300	\$748,293
3	261X0420-6,13	"F" FRAME	4	3.1	\$142,500	\$32,355	\$313,107	\$487,962
3	261X0420-7	"F" FRAME	1	7.2	\$309,900	\$74,088	\$169,755	\$553,743
3	261X0420-8	"F" FRAME	2	2.3	\$93,900	\$24,512	\$434,132	\$552,544
3	261X0420-9	"F" FRAME	1	7.2	\$309,900	\$74,088	\$169,755	\$553,743
3	261X0420-10,12	"F" FRAME	2	11.7	\$531,600	\$120,393	\$189,494	\$841,487
3	261X0420-11	"F" FRAME	1	13.2	\$485,700	\$131,947	\$142,145	\$759,792
4	261X0421-1	PANEL BOND ASSY.	1	12.0	\$4,268,400	\$238,500	\$10,589,192	\$15,096,092
6	261X0419-1,3	LWR SILL OUT CHRD	2	5.2	\$138,000	\$97,344	\$113,732	\$349,076
6	261X0419-2	LWR SILL OUT CHRD	1	8.0	\$165,000	\$155,520	\$48,244	\$368,764
6	261X0419-4,6	UPP SILL OUT CHRD	2	5.1	\$136,000	\$103,219	\$113,732	\$352,951
6	261X0419-5	UPP SILL OUT CHRD	1	8.1	\$167,000	\$157,464	\$48,244	\$372,708
6	261X0419-7,8	LWR SILL INR CHRD	2	3.1	\$92,000	\$84,226	\$57,885	\$234,111
6	261X0419-9,10	EDGE FRAME	2	73.5	\$1,290,900	\$515,970	\$401,774	\$2,208,644
6	261X0419-11 TO 26	INTERCOSTALS	16	12.0	\$123,000	\$266,112	\$117,897	\$507,009
6	261X0419-27,28	SHEAR TIES	2	0.4	\$9,000	\$5,654	\$17,119	\$31,773
6	261X0419-29 TO 36	BACKUP FITTING	16	5.7	\$12,226	\$82,696	\$39,271	\$134,193
6	261X0419-37,38	AUX FRAME	2	31.4	\$756,000	\$307,989	\$401,774	\$1,465,763
6	261X0418-3	FRAME SPLICE	8	1.2	\$22,000	\$16,182	\$25,934	\$64,116
6	261X0418-6	SILL BRACKET	1	0.1	\$3,000	\$1,163	\$15,989	\$20,152
6	261X0418-7	SILL BRACKET	1	0.2	\$5,000	\$2,697	\$19,379	\$27,076
6	261X0418-8	SILL BRACKET	1	0.1	\$3,000	\$1,488	\$12,372	\$16,860
6	261X0418-9	SILL BRACKET	1	0.1	\$3,000	\$1,748	\$13,503	\$18,251
6	261X0418-10,11	AL FITTING	2	1.1	\$210,000	\$86,652	\$36,225	\$332,877
6	261X0418-12,13	AL FITTING	2	1.3	\$195,000	\$28,188	\$30,705	\$253,893
6	261X0418-14,15	AL FITTING	2	1.1	\$570,000	\$85,180	\$78,919	\$734,099
6	261X0418-16,17	AL FITTING	2	1.0	\$495,000	\$52,992	\$76,504	\$624,496
6	261X0418-18,19	AL HINGE FITTING	2	4.6	\$1,260,000	\$263,736	\$197,168	\$1,720,904
6	261X0418-4,5,21,22	FAIL-SAFE CHORDS	8	1.5	\$117,000	\$20,817	\$169,432	\$307,249
2	261X0418-23	STRINGER CLIPS	453	25.1	\$178,551	\$452,854	\$19,153	\$650,558
6	261X0418-20,24,25	STRG TERM. CLIPS	42	1.4	\$31,000	\$23,539	\$21,640	\$76,179
4	261X0418-905	FRAME INSTL.	1	12.7	\$5,238,000	\$949,517	\$1,962,571	\$8,150,088
4	261X0418-904	STRG CLIP INSTL.	1	8.0	\$2,419,500	\$642,927	\$0	\$3,062,427
10	261X0418-903	LWR SILL ASSY.	1	0.5	\$218,967	\$24,395	\$36,686	\$280,048
10	261X0418-902	UPR SILL ASSY.	1	0.5	\$219,452	\$26,612	\$36,686	\$282,750
10	261X0418-901	DOOR STRCT ASSY.	1	1.2	\$675,838	\$64,094	\$486,753	\$1,226,685

SIDE PANEL COST & WEIGHT BREAKDOWN
FAMILY B (SKIN-STRINGER) - GLOBAL EVALUATION BASELINE
(Weight in lbs per shipset, Cost in \$ per 300 shipsets)

C	PART #	DESCRIPTION	QTY	WEIGHT	RECURRING LABOR	RECURRING MATERIAL	NON RECURRING	TOTAL
10	261X0418-1	DOOR INSTL.	1	16.8	\$4,606,581	\$1,022,579	\$2,809,777	\$8,438,937
7	261X0422-9,10	PASS. FLOOR BEAM	20	273.1	\$416,703	\$2,920,805	\$26,387	\$3,363,895
7	261X0422-11,12,13	STANCHION	20	66.5	\$329,000	\$1,288,075	\$135,938	\$1,753,013
8	261X0422-14,15	AL CARGO FB SPLIC	20	13.7	\$2,625,000	\$1,320,739	\$73,830	\$4,019,569
8	261X0422-4,5	SPLICE PLATE	1	8.9	\$113,000	\$149,253	\$965,287	\$1,227,540
8	261X0422-7	FRAME SPLICE	20	11.2	\$333,000	\$94,349	\$415,365	\$842,714
8	261X0422-8	FRAME SPL. FILLER	20	1.6	\$39,000	\$27,108	\$4,830	\$70,938
8	261X0422-2,16	END FRAME	1	11.1	\$348,900	\$71,928	\$610,168	\$1,030,996
8	261X0422-3	FRAME SPLICE CHRD	20	3.9	\$263,000	\$27,378	\$112,479	\$402,857
8	261X0422-17 TO 31	AL STRG SPLICE FTG	23	11.1	\$3,046,500	\$274,349	\$706,301	\$4,027,150
8	261X0422-6,32:41	STRG SPLICE	17	3.2	\$49,000	\$37,665	\$116,669	\$203,334
8	261X0422-42	FRM SPLICE FILLER	20	0.7	\$24,000	\$10,512	\$0	\$34,512
8	261X0422-43:56	STRG SPLICE FILLER	57	2.4	\$46,000	\$34,992	\$0	\$80,992
2	261X0421-11,19	LONG. SPLICE STRG.	1	10.4	\$242,000	\$231,945	\$0	\$473,945
10	261X0422-908	LOWER LOBE ASSY.	1	0.0	\$470,100	\$0	\$0	\$470,100
10	261X0422-907	PASS. FB INSTL.	1	2.8	\$973,590	\$150,903	\$6,767	\$1,131,260
10	261X0422-906	PASS. STAN. INSTL.	1	3.1	\$1,264,537	\$179,288	\$13,533	\$1,457,358
10	261X0422-905	LONG. LAP SPLICE	1	5.4	\$1,109,000	\$306,295	\$5,085	\$1,420,380
10	261X0422-904	FRAME SPLICE INSTL	1	4.8	\$1,776,551	\$285,507	\$11,278	\$2,073,335
10	261X0422-903	CARGO FB INSTL.	1	9.6	\$2,356,485	\$524,627	\$0	\$2,881,112
10	261X0422-902	OVRRWG LNG INSTL	1	1.5	\$324,000	\$116,785	\$0	\$440,785
10	261X0422-901	46/47 SEC. JOIN	1	6.9	\$2,634,886	\$392,582	\$2,260,627	\$5,288,095
10	261X0422-1	46/45 SEC. JOIN	1	7.1	\$2,564,716	\$438,065	\$486,070	\$3,488,852
TOTAL				1984.4	\$62,968,583	\$31,070,107	\$31,410,059	\$125,448,749

SUMMARY

C	DESCRIPTION	WEIGHT	RECURRING LABOR	RECURRING MATERIAL	NON RECURRING	TOTAL
1	Skin	780.7	\$1,983,000	\$8,263,100	\$294,113	\$10,540,213
2	Stringers/Core	233.9	\$5,766,351	\$4,780,462	\$3,324,557	\$13,871,370
3	Frames	215.2	\$8,234,400	\$2,165,140	\$2,986,378	\$13,385,918
5	Window Belt	88.0	\$2,428,000	\$1,881,000	\$875,292	\$5,184,292
4	Skin Panel Ass'y	32.7	\$11,925,900	\$1,830,944	\$12,551,763	\$26,308,607
6	Door Reinforce. Fab	166.2	\$5,803,126	\$2,360,576	\$2,057,440	\$10,221,142
7	Pass. Floor Details	339.6	\$745,703	\$4,208,880	\$162,325	\$5,116,908
8	Splice Details Fab	67.9	\$6,887,400	\$2,048,273	\$3,004,929	\$11,940,602
10	Assembly	60.2	\$19,194,703	\$3,531,732	\$6,153,262	\$28,879,698
TOTAL		1984.4	\$62,968,583	\$31,070,107	\$31,410,059	\$125,448,749

SIDE PANEL COST & WEIGHT BREAKDOWN
FAMILY D (SANDWICH) - GLOBAL EVALUATION BASELINE
(Weight in lbs per shipset, Cost in \$ per 300 shipsets)

C	PART #	DESCRIPTION	QTY	WEIGHT	RECURRING LABOR	RECURRING MATERIAL	NON RECURRING	TOTAL
1	261X0415-3	OUTER SKIN	1	472.7	\$1,133,000	\$5,672,444	\$36,855	\$6,842,299
2	261X0415-8	CORE FAB	1	148.7	\$1,183,800	\$1,459,568	\$2,588	\$2,645,956
2	261X0415-5	CORE FAB	1	38.4	\$500,100	\$376,916	\$0	\$877,016
2	261X0415-6A	CORE FAB	1	15.6	\$352,500	\$300,004	\$2,588	\$655,092
2	261X0415-7A	CORE FAB	1	1.7	\$61,500	\$32,693	\$0	\$94,193
2	261X0415-20,21	LONG. EDGE BAND	2	24.7	\$1,315,000	\$173,394	\$752,979	\$2,241,373
2	261X0415-22 TO 26	CIRC. EDGE BAND	5	36.7	\$499,900	\$217,998	\$234,930	\$952,828
2	261X0415-16 TO 19	WINDOW CLOSEOUT	14	31.9	\$1,000,400	\$189,486	\$32,360	\$1,222,246
2	261X0415-10,11	PASS. CLOSEOUT	2	25.0	\$244,200	\$516,000	\$140,682	\$900,882
2	261X0415-9,12	PASS. CLOSEOUT	2	2.9	\$57,100	\$59,856	\$9,660	\$126,616
2	261X0415-13 TO 15	OWRWNG CLOSEOUT	3	2.9	\$64,900	\$59,856	\$75,171	\$199,927
2	261X0415-4	CORE PANEL ASSY.	1	9.0	\$3,746,800	\$47,250	\$1,959,174	\$5,753,224
2	261X0415-27,28	FLOOR TRUSS TIE	15	6.6	\$255,000	\$121,750	\$313,711	\$690,461
1	261X0415-3	INNER SKIN	1	443.7	\$949,200	\$5,353,763	\$36,855	\$6,339,818
3	261X0416-1 TO 5	FUSELAGE FRAMES	25	188.1	\$6,066,300	\$1,181,617	\$1,471,513	\$8,719,430
6	261X0416-6	AUX PASS FRAMES	2	26.7	\$527,700	\$178,783	\$252,920	\$959,403
4	261X0415-901,1	PANEL BOND ASSY.	1	97.9	\$5,000,400	\$1,945,763	\$11,031,137	\$17,977,300
6	261X0414-1,3	LWR SILL OUT CHRDR	2	5.3	\$140,000	\$99,216	\$113,732	\$352,948
6	261X0414-2	LWR SILL OUT CHRDR	1	8.3	\$170,000	\$161,352	\$48,244	\$379,596
6	261X0414-4,6	UPP SILL OUT CHRDR	2	5.3	\$140,000	\$107,265	\$113,732	\$360,997
6	261X0414-5	UPP SILL OUT CHRDR	1	8.4	\$172,000	\$163,296	\$48,244	\$383,540
6	261X0414-7,8	LWR SILL INR CHRDR	2	3.1	\$92,000	\$84,226	\$57,885	\$234,111
6	261X0414-9,10	EDGE FRAME	1	76.4	\$1,194,000	\$536,328	\$401,774	\$2,132,102
6	261X0419-11 TO 26	INTERCOSTALS	16	12.0	\$123,000	\$266,112	\$117,897	\$507,009
6	261X0419-27, 28	SHEAR TIES	2	0.4	\$9,000	\$5,654	\$17,119	\$31,773
6	261X0414-35 TO 42	BACKUP FITTING	16	5.7	\$72,000	\$82,696	\$39,271	\$193,967
6	261X0414-47	ANGLE	2	0.1	\$1,875	\$1,200	\$16,667	\$19,742
6	261X0413-3,4,5,21	FRAME SPLICE	8	1.7	\$130,000	\$16,708	\$201,791	\$348,499
6	261X0413-6	SILL BRACKET	1	0.1	\$3,000	\$1,163	\$15,989	\$20,152
6	261X0413-7	SILL BRACKET	1	0.2	\$5,000	\$2,697	\$19,379	\$27,076
6	261X0413-8	SILL BRACKET	1	0.1	\$3,000	\$1,488	\$12,372	\$16,860
6	261X0413-9	SILL BRACKET	1	0.1	\$3,000	\$1,748	\$13,503	\$18,251
6	261X0413-20	BRACKET	2	0.0	\$2,000	\$949	\$14,407	\$17,356
6	261X0413-10,11	AL FITTING	2	1.1	\$210,000	\$86,652	\$36,225	\$332,877
6	261X0413-12,13	AL FITTING	2	1.3	\$195,000	\$28,188	\$30,705	\$253,893
6	261X0413-14,15	AL FITTING	2	1.1	\$570,000	\$85,180	\$78,919	\$734,099
6	261X0413-16,17	AL FITTING	2	1.0	\$495,000	\$52,992	\$76,504	\$624,496
6	261X0413-18,19	AL HINGE FITTING	2	4.6	\$1,260,000	\$263,736	\$197,168	\$1,720,904
5	216X0417-1,2,3	WINDOW FRAME	14	47.7	\$0	\$1,678,000	\$0	\$1,678,000
5	261X0413-9061	F/G ISOLATION FAB	1	6.1	\$73,000	\$159,283	\$6,210	\$238,493
5	261X0413-906	F/G ISOLATION BOND	1	0.0	\$1,486,921	\$0	\$14,279	\$1,501,200
4	261X0413-905	WINDOW FRM INSTL.	1	6.7	\$3,023,112	\$329,978	\$30,307	\$3,383,397
10	261X0413-904	INTERCOSTAL ASSY.	0	0.0	\$0	\$0	\$0	\$0
10	261X0413-903	LWR SILL ASSY.	1	0.4	\$218,967	\$24,123	\$36,686	\$279,776
10	261X0413-902	UPR SILL ASSY.	1	0.4	\$219,452	\$25,768	\$36,686	\$281,906
10	261X0413-901	DOOR STRCT ASSY.	1	1.0	\$675,838	\$61,760	\$486,753	\$1,224,351
10	261X0413-1	DOOR INSTL.	1	14.9	\$4,320,971	\$987,843	\$2,809,777	\$8,118,591
7	261X0422-9,10	PASS. FLOOR BEAM	20	273.1	\$416,703	\$2,920,805	\$26,387	\$3,363,895
7	261X0412-11,12	STANCHION	20	65.9	\$351,000	\$1,288,075	\$135,938	\$1,775,013
8	261X0412-14	AL CARGO FB SPLIC	20	13.7	\$2,850,000	\$1,319,838	\$73,830	\$4,243,668
8	261X0412-4,5,6	SPLICE PLATE	2	21.3	\$210,000	\$357,201	\$888,492	\$1,455,693
8	261X0412-7	FRAME SPLICE	20	11.2	\$350,400	\$94,349	\$379,122	\$823,871
8	261X0412-8	FRAME SPL. FILLER	20	1.0	\$28,000	\$17,157	\$600,788	\$645,945
8	261X0412-2,16	END FRAME	1	15.5	\$313,800	\$100,440	\$752,551	\$1,166,791
8	261X0412-3	SHEAR TIES	2	1.6	\$118,000	\$17,107	\$323,463	\$458,570
10	261X0412-908	LOWER LOBE ASSY.	1	0.0	\$470,000	\$0	\$0	\$470,000

SIDE PANEL COST & WEIGHT BREAKDOWN
FAMILY D (SANDWICH) - GLOBAL EVALUATION BASELINE
(Weight in lbs per shipset, Cost in \$ per 300 shipsets)

C	PART #	DESCRIPTION	QTY	WEIGHT	RECURRING LABOR	RECURRING MATERIAL	NON RECURRING	TOTAL
10	261X0412-907	PASS. FB INSTL.	1	2.7	\$632,562	\$164,329	\$6,767	\$803,658
10	261X0412-906	PASS. STAN. INSTL.	1	3.2	\$911,959	\$187,441	\$13,533	\$1,112,933
10	261X0412-905	LONG. LAP SPLICE	1	22.1	\$2,385,335	\$1,213,670	\$5,085	\$3,604,089
10	261X0412-904	FRAME SPLICE	1	4.2	\$1,118,194	\$252,822	\$9,022	\$1,380,038
10	261X0412-903	CARGO FB INSTL.	1	9.8	\$1,975,627	\$552,151	\$0	\$2,527,778
10	261X0412-902	OVRWNG LNG INSTL	1	1.4	\$173,296	\$109,163	\$0	\$282,459
10	261X0412-901	46/47 SEC. JOIN	1	7.1	\$1,684,457	\$402,441	\$2,260,135	\$4,347,033
10	261X0412-1	46/45 SEC. JOIN	1	6.9	\$1,684,457	\$387,391	\$485,578	\$2,557,427
TOTAL				2247.4	\$53,639,727	\$32,587,122	\$27,406,036	\$113,632,885

SUMMARY

C	DESCRIPTION	WEIGHT	RECURRING LABOR	RECURRING MATERIAL	NON RECURRING	TOTAL
1	Skin	916.4	\$2,082,200	\$11,026,207	\$73,711	\$13,182,118
2	Stringers/Core	344.1	\$9,281,200	\$3,554,771	\$3,523,842	\$16,359,813
3	Frames	188.1	\$6,066,300	\$1,181,617	\$1,471,513	\$8,719,430
5	Window Belt	53.8	\$1,559,921	\$1,837,283	\$20,489	\$3,417,693
4	Skin Panel Ass'y	104.6	\$8,023,512	\$2,275,741	\$11,061,443	\$21,360,696
6	Door Reinforce. Fab	163.0	\$5,517,575	\$2,227,629	\$1,924,444	\$9,669,648
7	Pass. Floor Details	339.0	\$767,703	\$4,208,880	\$162,325	\$5,138,908
8	Splice Details Fab	64.3	\$3,870,200	\$1,906,092	\$3,018,246	\$8,794,538
10	Assembly	74.1	\$16,471,116	\$4,368,902	\$6,150,023	\$26,990,040
TOTAL		2247.4	\$53,639,727	\$32,587,122	\$27,406,036	\$113,632,885

SIDE PANEL COST & WEIGHT BREAKDOWN
FAMILY B (SKIN-STRINGER) - GLOBAL OPTIMIZATION
(Weight in lbs per shipset, Cost in \$ per 300 shipsets)

C	PART #	DESCRIPTION	QTY	WEIGHT	RECURRING LABOR	RECURRING MATERIAL	NON RECURRING	TOTAL
1	261X0421-2	SKIN	1	797.4	\$2,090,000	\$8,589,298	\$294,113	\$10,973,411
2	261X0421-11	"J" STRINGER	2	12.9	\$468,000	\$261,922	\$249,016	\$978,938
2	261X0421-12	"J" STRINGER	1	4.2	\$165,000	\$85,882	\$135,772	\$386,654
2	261X0421-13	"J" STRINGER	7	4.0	\$56,878	\$69,682	\$39,271	\$165,831
2	261X0421-14	"J" STRINGER	4	19.1	\$383,719	\$332,731	\$39,271	\$755,721
2	261X0421-15	"J" STRINGER	2	7.8	\$319,000	\$158,371	\$249,016	\$726,387
2	261X0421-16	"J" STRINGER	2	19.8	\$468,000	\$407,722	\$249,016	\$1,124,738
2	261X0421-17	"J" STRINGER	2	12.2	\$468,000	\$251,222	\$249,016	\$968,238
2	261X0421-18	"J" STRINGER	7	64.6	\$909,744	\$1,125,364	\$39,271	\$2,074,379
2	261X0421-19	"J" STRINGER	3	43.4	\$672,000	\$998,784	\$356,629	\$2,027,413
2	261X0421-20	"J" STRINGER	1	6.1	\$162,000	\$145,814	\$135,772	\$443,586
2	261X0421-21	"J" STRINGER	1	1.2	\$8,125	\$20,905	\$39,271	\$68,301
5	417-1A	2D BRAID WIN. FRM.	14	28.0	\$3,025,500	\$598,500	\$295,000	\$3,919,000
3	261X0420-1	"F" FRAME	9	119.2	\$4,372,200	\$1,191,523	\$1,045,255	\$6,608,978
3	261X0420-2	"F" FRAME	1	13.2	\$485,700	\$131,947	\$142,145	\$759,792
3	261X0420-3	"F" FRAME	1	13.2	\$485,700	\$131,947	\$142,145	\$759,792
3	261X0420-4	"F" FRAME	1	13.2	\$485,700	\$131,947	\$142,145	\$759,792
3	261X0420-5	"F" FRAME	2	11.7	\$531,600	\$120,393	\$96,300	\$748,293
3	261X0420-6,13	"F" FRAME	4	3.1	\$142,500	\$32,355	\$313,107	\$487,962
3	261X0420-7	"F" FRAME	1	7.2	\$309,900	\$74,088	\$169,755	\$553,743
3	261X0420-8	"F" FRAME	2	2.3	\$93,900	\$24,512	\$434,132	\$552,544
3	261X0420-9	"F" FRAME	1	7.2	\$309,900	\$74,088	\$169,755	\$553,743
3	261X0420-10,12	"F" FRAME	2	11.7	\$531,600	\$120,393	\$189,494	\$841,487
3	261X0420-11	"F" FRAME	1	13.2	\$485,700	\$131,947	\$142,145	\$759,792
4	261X0421-1	PANEL BOND ASSY.	1	12.0	\$4,226,400	\$238,500	\$10,589,192	\$15,054,092
6	261X0419-1,3	LWR SILL OUT CHRD	2	5.2	\$138,000	\$97,344	\$113,732	\$349,076
6	261X0419-2	LWR SILL OUT CHRD	1	8.0	\$165,000	\$155,520	\$48,244	\$368,764
6	261X0419-4,6	UPP SILL OUT CHRD	2	5.1	\$136,000	\$103,219	\$113,732	\$352,951
6	261X0419-5	UPP SILL OUT CHRD	1	8.1	\$167,000	\$157,464	\$48,244	\$372,708
6	261X0419-7,8	LWR SILL INR CHRD	2	3.1	\$92,000	\$84,226	\$57,885	\$234,111
6	261X0419-9,10	EDGE FRAME	2	73.5	\$1,290,900	\$515,970	\$401,774	\$2,208,644
6	261X0419-11 TO 26	INTERCOSTALS	16	12.0	\$123,000	\$266,112	\$117,897	\$507,009
6	261X0419-27,28	SHEAR TIES	2	0.4	\$9,000	\$5,654	\$17,119	\$31,773
6	261X0419-29 TO 36	BACKUP FITTING	16	5.7	\$12,226	\$82,696	\$39,271	\$134,193
6	261X0419-37,38	AUX FRAME	2	31.4	\$756,000	\$307,989	\$401,774	\$1,465,763
6	261X0418-3	FRAME SPLICE	8	1.2	\$22,000	\$16,182	\$25,934	\$64,116
6	261X0418-6	SILL BRACKET	1	0.1	\$3,000	\$1,163	\$15,989	\$20,152
6	261X0418-7	SILL BRACKET	1	0.2	\$5,000	\$2,697	\$19,379	\$27,076
6	261X0418-8	SILL BRACKET	1	0.1	\$3,000	\$1,488	\$12,372	\$16,860
6	261X0418-9	SILL BRACKET	1	0.1	\$3,000	\$1,748	\$13,503	\$18,251
6	261X0418-10,11	AL FITTING	2	1.1	\$210,000	\$86,652	\$36,225	\$332,877
6	261X0418-12,13	AL FITTING	2	1.3	\$195,000	\$28,188	\$30,705	\$253,893
6	261X0418-14,15	AL FITTING	2	1.1	\$570,000	\$85,180	\$78,919	\$734,099
6	261X0418-16,17	AL FITTING	2	1.0	\$495,000	\$52,992	\$76,504	\$624,496
6	261X0418-18,19	AL HINGE FITTING	2	4.6	\$1,260,000	\$263,736	\$197,168	\$1,720,904
6	261X0418-4,5,21,22	FAIL-SAFE CHORDS	8	1.5	\$117,000	\$20,817	\$169,432	\$307,249
2	261X0418-23	STRINGER CLIPS	434	24.0	\$171,062	\$433,860	\$19,153	\$624,076
6	261X0418-20,24,25	STRG TERM. CLIPS	42	1.4	\$31,000	\$23,539	\$21,640	\$76,179
4	261X0418-905	FRAME INSTL.	1	12.7	\$5,238,000	\$949,517	\$1,962,571	\$8,150,088
4	261X0418-904	STRG CLIP INSTL.	1	7.7	\$2,318,020	\$615,961	\$0	\$2,933,981
10	261X0418-903	LWR SILL ASSY.	1	0.5	\$218,967	\$24,395	\$36,686	\$280,048
10	261X0418-902	UPR SILL ASSY.	1	0.5	\$219,452	\$26,612	\$36,686	\$282,750
10	261X0418-901	DOOR STRCT ASSY.	1	1.2	\$675,838	\$64,094	\$486,753	\$1,226,685

SIDE PANEL COST & WEIGHT BREAKDOWN
FAMILY B (SKIN-STRINGER) - GLOBAL OPTIMIZATION
(Weight in lbs per shipset, Cost in \$ per 300 shipsets)

C	PART #	DESCRIPTION	QTY	WEIGHT	RECURRING LABOR	RECURRING MATERIAL	NON RECURRING	TOTAL
10	261X0418-1	DOOR INSTL.	1	16.8	\$4,606,581	\$1,022,579	\$2,809,777	\$8,438,937
7	261X0422-9,10	PASS. FLOOR BEAM	20	273.1	\$416,703	\$2,920,805	\$26,387	\$3,363,895
7	261X0422-11,12,13	STANCHION	20	66.5	\$329,000	\$1,288,075	\$135,938	\$1,753,013
8	261X0422-14,15	AL CARGO FB SPLIC	20	13.7	\$2,625,000	\$1,320,739	\$73,830	\$4,019,569
8	261X0422-4,5	SPLICE PLATE	1	8.9	\$113,000	\$149,253	\$965,287	\$1,227,540
8	261X0422-7	FRAME SPLICE	20	11.2	\$333,000	\$94,349	\$415,365	\$842,714
8	261X0422-8	FRAME SPL. FILLER	20	1.6	\$39,000	\$27,108	\$4,830	\$70,938
8	261X0422-2,16	END FRAME	1	11.1	\$348,900	\$71,928	\$610,168	\$1,030,996
8	261X0422-3	FRAME SPLICE CHRDL	20	3.9	\$263,000	\$27,378	\$112,479	\$402,857
8	261X0422-17 TO 31	AL STRG SPLICE FTG	21	10.1	\$2,781,587	\$250,493	\$644,884	\$3,676,963
8	261X0422-6,32:41	STRG SPLICE	16	3.1	\$46,118	\$35,449	\$116,669	\$198,237
8	261X0422-42	FRM SPLICE FILLER	20	0.7	\$24,000	\$10,512	\$0	\$34,512
8	261X0422-43:56	STRG SPLICE FILLER	57	2.4	\$46,000	\$34,992	\$0	\$80,992
8	261X0421-11,19	LONG. SPLICE STRG.	1	10.4	\$242,000	\$231,945	\$0	\$473,945
10	261X0422-908	LOWER LOBE ASSY.	1	0.0	\$470,100	\$0	\$0	\$470,100
10	261X0422-907	PASS. FB INSTL.	1	2.8	\$973,590	\$150,903	\$6,767	\$1,131,260
10	261X0422-906	PASS. STAN. INSTL.	1	3.1	\$1,264,537	\$179,288	\$13,533	\$1,457,358
10	261X0422-905	LONG. LAP SPLICE	1	5.4	\$1,109,000	\$306,295	\$5,085	\$1,420,380
10	261X0422-904	FRAME SPLICE INSTL	1	4.8	\$1,776,551	\$285,507	\$11,278	\$2,073,335
10	261X0422-903	CARGO FB INSTL.	1	9.6	\$2,356,485	\$524,627	\$0	\$2,881,112
10	261X0422-902	OVRRNG LNG INSTL	1	1.5	\$324,000	\$116,785	\$0	\$440,785
10	261X0422-901	46/47 SEC. JOIN	1	6.8	\$2,585,086	\$378,042	\$2,260,627	\$5,223,755
10	261X0422-1	46/45 SEC. JOIN	1	7.0	\$2,514,916	\$421,840	\$486,070	\$3,422,827
TOTAL				1935.2	\$61,889,385	\$29,773,744	\$29,244,268	\$120,907,398

SUMMARY

C	DESCRIPTION	WEIGHT	RECURRING LABOR	RECURRING MATERIAL	NON RECURRING	TOTAL
1	Skin	797.4	\$2,090,000	\$8,589,298	\$294,113	\$10,973,411
2	Stringers/Core	219.3	\$4,251,528	\$4,292,259	\$1,800,476	\$10,344,263
3	Frames	215.2	\$8,234,400	\$2,165,140	\$2,986,378	\$13,385,918
5	Window Belt	28.0	\$3,025,500	\$598,500	\$295,000	\$3,919,000
4	Skin Panel Ass'y	32.4	\$11,782,420	\$1,803,978	\$12,551,763	\$26,138,160
6	Door Reinforce. Fab	166.2	\$5,803,126	\$2,360,576	\$2,057,440	\$10,221,142
7	Pass. Floor Details	339.6	\$745,703	\$4,208,880	\$162,325	\$5,116,908
8	Splice Details Fab	77.1	\$6,861,605	\$2,254,146	\$2,943,512	\$12,059,262
10	Assembly	60.0	\$19,095,103	\$3,500,967	\$6,153,262	\$28,749,333
TOTAL		1935.2	\$61,889,385	\$29,773,744	\$29,244,268	\$120,907,398

SIDE PANEL COST & WEIGHT BREAKDOWN

FAMILY D (SANDWICH) - GLOBAL OPTIMIZATION

(Weight in lbs per shipset, Cost in \$ per 300 shipsets)

C	PART #	DESCRIPTION	QTY	WEIGHT	RECURRING LABOR	RECURRING MATERIAL	NON RECURRING	TOTAL
1	261X0415-3	OUTER SKIN	1	472.7	\$1,133,000	\$5,672,444	\$36,855	\$6,842,299
2	261X0415-8	CORE FAB	1	83.6	\$1,183,800	\$2,919,136	\$2,588	\$4,105,524
2	261X0415-5	CORE FAB	1	21.6	\$500,100	\$753,832	\$0	\$1,253,932
2	261X0415-6A	CORE FAB	1	0.0	\$34,500	\$104,500	\$2,588	\$141,588
2	261X0415-7A	CORE FAB	1	0.0	\$34,500	\$104,500	\$0	\$139,000
2	261X0415-20,21	LONG. EDGE BAND	2	24.7	\$1,315,000	\$173,394	\$752,979	\$2,241,373
2	261X0415-22 TO 26	CIRC. EDGE BAND	5	6.1	\$38,000	\$117,000	\$234,930	\$389,930
5	261X0415-K,L,M	FRM ATTACH "T"	45	20.5	\$1,438,000	\$207,000	\$323,000	\$1,968,000
2	261X0415-10,11	PASS. CLOSEOUT	2	12.9	\$191,000	\$243,000	\$140,682	\$574,682
2	261X0415-9,12	PASS. CLOSEOUT	2	2.9	\$57,100	\$59,856	\$9,660	\$126,616
2	261X0415-13 TO 15	OVRWNG CLOSEOUT	3	2.9	\$64,900	\$59,856	\$75,171	\$199,927
2	261X0415-4	CORE PANEL ASSY.	1	9.0	\$4,060,800	\$668,000	\$214,531	\$4,943,331
2	261X0415-27,28	FLOOR TRUSS TIE	15	6.6	\$255,000	\$121,750	\$313,711	\$690,461
1	261X0415-3	INNER SKIN	1	443.7	\$949,200	\$5,353,763	\$36,855	\$6,339,818
3	261X0416-1 TO 5	FUSELAGE FRAMES	25	188.1	\$6,066,300	\$1,181,617	\$1,471,513	\$8,719,430
6	261X0416-6	AUX PASS FRAMES	2	26.7	\$527,700	\$178,783	\$252,920	\$959,403
4	261X0415-901,1	PANEL BOND ASSY.	1	66.7	\$5,100,000	\$621,000	\$11,031,137	\$16,752,137
6	261X0414-1,3	LWR SILL OUT CHRDR	2	5.3	\$140,000	\$99,216	\$113,732	\$352,948
6	261X0414-2	LWR SILL OUT CHRDR	1	8.3	\$170,000	\$161,352	\$48,244	\$379,596
6	261X0414-4,6	UPP SILL OUT CHRDR	2	5.3	\$140,000	\$107,265	\$113,732	\$360,997
6	261X0414-5	UPP SILL OUT CHRDR	1	8.4	\$172,000	\$163,296	\$48,244	\$383,540
6	261X0414-7,8	LWR SILL INR CHRDR	2	3.1	\$92,000	\$84,226	\$57,885	\$234,111
6	261X0414-9,10	EDGE FRAME	1	76.4	\$1,194,000	\$536,328	\$401,774	\$2,132,102
6	261X0419-11 TO 26	INTERCOSTALS	16	12.0	\$123,000	\$266,112	\$117,897	\$507,009
6	261X0419-27,28	SHEAR TIES	2	0.4	\$9,000	\$5,654	\$17,119	\$31,773
6	261X0414-35 TO 42	BACKUP FITTING	16	5.7	\$72,000	\$82,696	\$39,271	\$193,967
6	261X0414-47	ANGLE	2	0.1	\$1,875	\$1,200	\$16,667	\$19,742
6	261X0413-3,4,5,21	FRAME SPLICE	8	1.7	\$130,000	\$16,708	\$201,791	\$348,499
6	261X0413-6	SILL BRACKET	1	0.1	\$3,000	\$1,163	\$15,989	\$20,152
6	261X0413-7	SILL BRACKET	1	0.2	\$5,000	\$2,697	\$19,379	\$27,076
6	261X0413-8	SILL BRACKET	1	0.1	\$3,000	\$1,488	\$12,372	\$16,860
6	261X0413-9	SILL BRACKET	1	0.1	\$3,000	\$1,748	\$13,503	\$18,251
6	261X0413-20	BRACKET	2	0.0	\$2,000	\$949	\$14,407	\$17,356
6	261X0413-10,11	AL FITTING	2	1.1	\$210,000	\$86,652	\$36,225	\$332,877
6	261X0413-12,13	AL FITTING	2	1.3	\$195,000	\$28,188	\$30,705	\$253,893
6	261X0413-14,15	AL FITTING	2	1.1	\$570,000	\$85,180	\$78,919	\$734,099
6	261X0413-16,17	AL FITTING	2	1.0	\$495,000	\$52,992	\$76,504	\$624,496
6	261X0413-18,19	AL HINGE FITTING	2	4.6	\$1,260,000	\$263,736	\$197,168	\$1,720,904
5	417-1A	2D BRAID WIN. FRM.	14	28.0	\$3,025,500	\$598,500	\$295,000	\$3,919,000
5	261X0413-9061	F/G ISOLATION FAB	0	0.0	\$0	\$0	\$0	\$0
5	261X0413-906	F/G ISOLATION BOND	0	0.0	\$0	\$0	\$0	\$0
4	261X0413-905	WINDOW FRM INSTL.	0	0.0	\$0	\$0	\$0	\$0
10	261X0413-904	INTERCOSTAL ASSY.	0	0.0	\$0	\$0	\$0	\$0
10	261X0413-903	LWR SILL ASSY.	1	0.4	\$218,967	\$24,123	\$36,686	\$279,776
10	261X0413-902	UPR SILL ASSY.	1	0.4	\$219,452	\$25,768	\$36,686	\$281,906
10	261X0413-901	DOOR STRCT ASSY.	1	1.0	\$675,838	\$61,760	\$486,753	\$1,224,351
10	261X0413-1	DOOR INSTL.	1	14.9	\$4,320,971	\$987,843	\$2,809,777	\$8,118,591
7	261X0422-9,10	PASS. FLOOR BEAM	20	273.1	\$416,703	\$2,920,805	\$26,387	\$3,363,895
7	261X0412-11,12	STANCHION	20	65.9	\$351,000	\$1,288,075	\$135,938	\$1,775,013
8	261X0412-14	AL CARGO FB SPLIC	20	13.7	\$2,850,000	\$1,319,838	\$73,830	\$4,243,668
8	261X0412-4,5,6	SPLICE PLATE	2	10.7	\$105,000	\$177,000	\$888,492	\$1,170,492
8	261X0412-7	FRAME SPLICE	20	11.2	\$350,400	\$94,349	\$379,122	\$823,871
8	261X0412-8	FRAME SPL. FILLER	20	1.0	\$28,000	\$17,157	\$600,788	\$645,945
8	261X0412-2,16	END FRAME	1	15.5	\$313,800	\$100,440	\$752,551	\$1,166,791
8	261X0412-3	SHEAR TIES	2	1.6	\$118,000	\$17,107	\$323,463	\$458,570
10	261X0412-908	LOWER LOBE ASSY.	1	0.0	\$470,000	\$0	\$0	\$470,000

SIDE PANEL COST & WEIGHT BREAKDOWN

FAMILY D (SANDWICH) - GLOBAL OPTIMIZATION

(Weight in lbs per shipset, Cost in \$ per 300 shipsets)

C	PART #	DESCRIPTION	QTY	WEIGHT	RECURRING LABOR	RECURRING MATERIAL	NON RECURRING	TOTAL
10	261X0412-907	PASS. FB INSTL.	1	2.7	\$632,562	\$164,329	\$6,767	\$803,658
10	261X0412-906	PASS. STAN. INSTL.	1	3.2	\$911,959	\$187,441	\$13,533	\$1,112,933
10	261X0412-905	LONG. LAP SPLICE	1	22.1	\$2,385,335	\$1,213,670	\$5,085	\$3,604,089
10	261X0412-904	FRAME SPLICE	1	4.2	\$1,118,194	\$252,822	\$9,022	\$1,380,038
10	261X0412-903	CARGO FB INSTL.	1	9.8	\$1,975,627	\$552,151	\$0	\$2,527,778
10	261X0412-902	OVRWNG LNG INSTL	1	1.4	\$173,296	\$109,163	\$0	\$282,459
10	261X0412-901	46/47 SEC. JOIN	1	7.1	\$1,684,457	\$402,441	\$2,260,135	\$4,347,033
10	261X0412-1	46/45 SEC. JOIN	1	6.9	\$1,684,457	\$387,391	\$485,578	\$2,557,427
TOTAL				2019.8	\$51,968,294	\$31,490,450	\$26,196,238	\$109,654,982

SUMMARY

C	DESCRIPTION	WEIGHT	RECURRING LABOR	RECURRING MATERIAL	NON RECURRING	TOTAL
1	Skin	916.4	\$2,082,200	\$11,026,207	\$73,711	\$13,182,118
2	Stringers/Core	170.3	\$7,734,700	\$5,324,824	\$1,746,840	\$14,525,776
3	Frames	188.1	\$6,066,300	\$1,181,617	\$1,471,513	\$8,719,430
5	Window Belt	48.5	\$4,463,500	\$805,500	\$618,000	\$16,752,137
4	Skin Panel Ass'y	66.7	\$5,100,000	\$621,000	\$11,031,137	\$6,167,588
6	Door Reinforce. Fab	163.0	\$5,517,575	\$2,227,629	\$1,924,444	\$9,669,648
7	Pass. Floor Details	339.0	\$767,703	\$4,208,880	\$162,325	\$5,138,908
8	Splice Details Fab	53.7	\$3,765,200	\$1,725,891	\$3,018,246	\$4,265,669
10	Assembly	74.1	\$16,471,116	\$4,368,902	\$6,150,023	\$17,085,415
TOTAL		2019.8	\$51,968,294	\$31,490,450	\$26,196,238	\$109,654,982

SIDE PANEL COST & WEIGHT BREAKDOWN
FAMILY B (SKIN-STRINGER) - LOCAL OPTIMIZATION POTENTIAL
(Weight in lbs per shipset, Cost in \$ per 300 shipsets)

C	PART #	DESCRIPTION	QTY	WEIGHT	RECURRING LABOR	RECURRING MATERIAL	NON RECURRING	TOTAL
1	261X0421-2	SKIN	2	897.4	\$2,090,000	\$9,666,461	\$294,113	\$12,050,574
2	261X0421-11	"HAT" STRINGER	1	12.9	\$119,400	\$261,922	\$124,508	\$505,830
2	261X0421-12	"HAT" STRINGER	1	4.2	\$119,400	\$85,882	\$135,772	\$341,054
2	261X0421-13	"HAT" STRINGER	7	4.0	\$34,127	\$69,682	\$135,772	\$239,581
2	261X0421-14	"HAT" STRINGER	2	19.1	\$238,800	\$332,731	\$237,753	\$809,284
2	261X0421-15	"HAT" STRINGER	1	7.8	\$119,400	\$158,371	\$124,508	\$402,279
2	261X0421-16	"HAT" STRINGER	2	19.8	\$238,800	\$407,722	\$249,016	\$895,538
2	261X0421-17	"HAT" STRINGER	1	12.2	\$119,400	\$251,222	\$124,508	\$495,130
2	261X0421-18	"HAT" STRINGER	5	64.6	\$597,000	\$1,125,364	\$582,314	\$2,304,678
2	261X0421-19	"HAT" STRINGER	2	43.4	\$238,800	\$998,784	\$475,506	\$1,713,090
2	261X0421-20	"HAT" STRINGER	1	6.1	\$119,400	\$145,814	\$67,886	\$333,100
2	261X0421-21	"HAT" STRINGER	1	1.2	\$4,875	\$20,905	\$135,772	\$161,552
5	417-1A	2D BRAID WIN. FRM.	14	28.0	\$3,025,500	\$598,500	\$295,000	\$3,919,000
3	261X0420-1	"F" FRAME	9	119.2	\$4,372,200	\$1,191,523	\$1,045,255	\$6,608,978
3	261X0420-2	"F" FRAME	1	13.2	\$485,700	\$131,947	\$142,145	\$759,792
3	261X0420-3	"F" FRAME	1	13.2	\$485,700	\$131,947	\$142,145	\$759,792
3	261X0420-4	"F" FRAME	1	13.2	\$485,700	\$131,947	\$142,145	\$759,792
3	261X0420-5	"F" FRAME	2	11.7	\$531,600	\$120,393	\$96,300	\$748,293
3	261X0420-6,13	"F" FRAME	4	3.1	\$142,500	\$32,355	\$313,107	\$487,962
3	261X0420-7	"F" FRAME	1	7.2	\$309,900	\$74,088	\$169,755	\$553,743
3	261X0420-8	"F" FRAME	2	2.3	\$93,900	\$24,512	\$434,132	\$552,544
3	261X0420-9	"F" FRAME	1	7.2	\$309,900	\$74,088	\$169,755	\$553,743
3	261X0420-10,12	"F" FRAME	2	11.7	\$531,600	\$120,393	\$189,494	\$841,487
3	261X0420-11	"F" FRAME	1	13.2	\$485,700	\$131,947	\$142,145	\$759,792
4	261X0421-1	PANEL BOND ASSY.	1	12.0	\$3,920,400	\$238,500	\$10,589,192	\$14,748,092
6	261X0419-1,3	LWR SILL OUT CHRDR	2	5.2	\$138,000	\$97,344	\$113,732	\$349,076
6	261X0419-2	LWR SILL OUT CHRDR	1	8.0	\$165,000	\$155,520	\$48,244	\$368,764
6	261X0419-4,6	UPP SILL OUT CHRDR	2	5.1	\$136,000	\$103,219	\$113,732	\$352,951
6	261X0419-5	UPP SILL OUT CHRDR	1	8.1	\$167,000	\$157,464	\$48,244	\$372,708
6	261X0419-7,8	LWR SILL INR CHRDR	2	3.1	\$92,000	\$84,226	\$57,885	\$234,111
6	261X0419-9,10	EDGE FRAME	2	73.5	\$1,290,900	\$515,970	\$401,774	\$2,208,644
6	261X0419-11 TO 26	INTERCOSTALS	16	12.0	\$123,000	\$266,112	\$117,897	\$507,009
6	261X0419-27,28	SHEAR TIES	2	0.4	\$9,000	\$5,654	\$17,119	\$31,773
6	261X0419-29 TO 36	BACKUP FITTING	16	5.7	\$12,226	\$82,696	\$39,271	\$134,193
6	261X0419-37,38	AUX FRAME	2	31.4	\$756,000	\$307,989	\$401,774	\$1,465,763
6	261X0418-3	FRAME SPLICE	8	1.2	\$22,000	\$16,182	\$25,934	\$64,116
6	261X0418-6	SILL BRACKET	1	0.1	\$3,000	\$1,163	\$15,989	\$20,152
6	261X0418-7	SILL BRACKET	1	0.2	\$5,000	\$2,697	\$19,379	\$27,076
6	261X0418-8	SILL BRACKET	1	0.1	\$3,000	\$1,488	\$12,372	\$16,860
6	261X0418-9	SILL BRACKET	1	0.1	\$3,000	\$1,748	\$13,503	\$18,251
6	261X0418-10,11	AL FITTING	2	1.1	\$210,000	\$86,652	\$36,225	\$332,877
6	261X0418-12,13	AL FITTING	2	1.3	\$195,000	\$28,188	\$30,705	\$253,893
6	261X0418-14,15	AL FITTING	2	1.1	\$570,000	\$85,180	\$78,919	\$734,099
6	261X0418-16,17	AL FITTING	2	1.0	\$495,000	\$52,992	\$76,504	\$624,496
6	261X0418-18,19	AL HINGE FITTING	2	4.6	\$1,260,000	\$263,736	\$197,168	\$1,720,904
6	261X0418-4,5,21,22	FAIL-SAFE CHORDS	8	1.5	\$117,000	\$20,817	\$169,432	\$307,249
2	261X0418-23	STRINGER CLIPS	0	0.0	\$0	\$0	\$0	\$0
6	261X0418-20,24,25	STRG TERM. CLIPS	42	1.4	\$31,000	\$23,539	\$21,640	\$76,179
4	261X0418-905	FRAME INSTL.	1	18.8	\$2,108,609	\$1,357,325	\$1,962,571	\$5,428,505
4	261X0418-904	STRG CLIP INSTL.	0	0.0	\$0	\$0	\$0	\$0
10	261X0418-903	LWR SILL ASSY.	1	0.5	\$67,442	\$24,395	\$36,686	\$128,523
10	261X0418-902	UPR SILL ASSY.	1	0.5	\$68,250	\$26,612	\$36,686	\$131,548
10	261X0418-901	DOOR STRCT ASSY.	1	1.2	\$275,742	\$64,094	\$486,753	\$826,589

SIDE PANEL COST & WEIGHT BREAKDOWN
FAMILY B (SKIN-STRINGER) - LOCAL OPTIMIZATION POTENTIAL
(Weight in lbs per shipset, Cost in \$ per 300 shipsets)

C	PART #	DESCRIPTION	QTY	WEIGHT	RECURRING LABOR	RECURRING MATERIAL	NON RECURRING	TOTAL
10	261X0418-1	DOOR INSTL.	1	16.8	\$1,229,957	\$1,022,579	\$2,809,777	\$5,062,313
7	261X0422-9,10	PASS. FLOOR BEAM	20	273.1	\$416,703	\$2,920,805	\$26,387	\$3,363,895
7	261X0422-11,12,13	STANCHION	20	66.5	\$329,000	\$1,288,075	\$135,938	\$1,753,013
8	261X0422-14,15	AL CARGO FB SPLIC	20	13.7	\$2,625,000	\$1,320,739	\$73,830	\$4,019,569
8	261X0422-4,5	SPLICE PLATE	1	8.9	\$113,000	\$149,253	\$965,287	\$1,227,540
8	261X0422-7	FRAME SPLICE	20	11.2	\$333,000	\$94,349	\$415,365	\$842,714
8	261X0422-8	FRAME SPL. FILLER	20	1.6	\$39,000	\$27,108	\$4,830	\$70,938
8	261X0422-2,16	END FRAME	1	11.1	\$348,900	\$71,928	\$610,168	\$1,030,996
8	261X0422-3	FRAME SPLICE CHRD	20	3.9	\$263,000	\$27,378	\$112,479	\$402,857
8	261X0422-17 TO 31	AL STRG SPLICE FTG	14	6.7	\$2,028,969	\$182,716	\$470,397	\$2,682,082
8	261X0422-6,32:41	STRG SPLICE	12	2.3	\$36,750	\$28,249	\$87,502	\$152,501
8	261X0422-42	FRM SPLICE FILLER	20	0.7	\$24,000	\$10,512	\$0	\$34,512
8	261X0422-43:56	STRG SPLICE FILLER	0	0.0	\$0	\$0	\$0	\$0
8	261X0421-11,19	LONG. SPLICE STRG.	1	10.4	\$242,000	\$231,945	\$0	\$473,945
10	261X0422-908	LOWER LOBE ASSY.	1	0.0	\$470,100	\$0	\$0	\$470,100
10	261X0422-907	PASS. FB INSTL.	1	2.8	\$331,021	\$150,903	\$6,767	\$488,690
10	261X0422-906	PASS. STAN. INSTL.	1	3.1	\$429,942	\$179,288	\$13,533	\$622,764
10	261X0422-905	LONG. LAP SPLICE	1	5.4	\$624,367	\$306,295	\$5,085	\$935,747
10	261X0422-904	FRAME SPLICE INSTL	1	4.8	\$639,558	\$285,507	\$11,278	\$936,343
10	261X0422-903	CARGO FB INSTL.	1	9.6	\$801,205	\$524,627	\$0	\$1,325,832
10	261X0422-902	OVRWNG LNG INSTL	1	1.5	\$110,160	\$116,785	\$0	\$226,945
10	261X0422-901	46/47 SEC. JOIN	1	6.8	\$942,446	\$378,042	\$2,260,627	\$3,581,115
10	261X0422-1	46/45 SEC. JOIN	1	7.0	\$917,348	\$421,840	\$486,070	\$1,825,259
TOTAL				2003.0	\$40,838,297	\$30,098,926	\$29,633,453	\$100,570,675

SUMMARY

C	DESCRIPTION	WEIGHT	RECURRING LABOR	RECURRING MATERIAL	NON RECURRING	TOTAL
1	Skin	897.4	\$2,090,000	\$9,666,461	\$294,113	\$12,050,574
2	Stringers/Core	195.3	\$1,949,402	\$3,858,399	\$2,393,315	\$8,201,116
3	Frames	215.2	\$8,234,400	\$2,165,140	\$2,986,378	\$13,385,918
5	Window Belt	28.0	\$3,025,500	\$598,500	\$295,000	\$14,748,092
4	Skin Panel Ass'y	30.8	\$6,029,009	\$1,595,825	\$12,551,763	\$3,919,000
6	Door Reinforce. Fab	166.2	\$5,803,126	\$2,360,576	\$2,057,440	\$10,221,142
7	Pass. Floor Details	339.6	\$745,703	\$4,208,880	\$162,325	\$5,116,908
8	Splice Details Fab	70.5	\$6,053,619	\$2,144,177	\$2,739,857	\$6,918,084
10	Assembly	60.0	\$6,907,538	\$3,500,967	\$6,153,262	\$10,412,794
TOTAL		2003.0	\$40,838,297	\$30,098,926	\$29,633,453	\$100,570,675

SIDE PANEL COST & WEIGHT BREAKDOWN
FAMILY D (SANDWICH) - LOCAL OPTIMIZATION POTENTIAL
(Weight in lbs per shipset, Cost in \$ per 300 shipsets)

C	PART #	DESCRIPTION	QTY	WEIGHT	RECURRING LABOR	RECURRING MATERIAL	NON RECURRING	TOTAL
1	261X0415-3	OUTER SKIN	1	418.7	\$908,609	\$4,839,957	\$36,855	\$5,785,421
2	261X0415-8	CORE FAB	1	83.6	\$1,183,800	\$2,919,136	\$2,588	\$4,105,524
2	261X0415-5	CORE FAB	1	21.6	\$500,100	\$753,832	\$0	\$1,253,932
2	261X0415-6A	CORE FAB	1	0.0	\$34,500	\$104,500	\$2,588	\$141,588
2	261X0415-7A	CORE FAB	1	0.0	\$34,500	\$104,500	\$0	\$139,000
2	261X0415-20,21	LONG. EDGE BAND	0	0.0	\$0	\$0	\$0	\$0
2	261X0415-22 TO 26	CIRC. EDGE BAND	5	6.1	\$38,000	\$117,000	\$234,930	\$389,930
5	261X0415-K,L,M	FRM ATTACH "T"	23	10.3	\$719,000	\$103,500	\$161,500	\$984,000
2	261X0415-10,11	PASS. CLOSEOUT	2	12.9	\$191,000	\$243,000	\$140,682	\$574,682
2	261X0415-9,12	PASS. CLOSEOUT	2	2.9	\$57,100	\$59,856	\$9,660	\$126,616
2	261X0415-13 TO 15	OVRRNG CLOSEOUT	3	2.9	\$64,900	\$59,856	\$75,171	\$199,927
2	261X0415-4	CORE PANEL ASSY.	1	9.0	\$4,060,800	\$668,000	\$214,531	\$4,943,331
2	261X0415-27,28	FLOOR TRUSS TIE	15	6.6	\$255,000	\$121,750	\$313,711	\$690,461
1	261X0415-3	INNER SKIN	1	403.1	\$776,700	\$4,668,638	\$36,855	\$5,482,193
3	261X0416-1 TO 5	FUSELAGE FRAMES	13	188.1	\$3,033,150	\$590,809	\$735,756	\$4,359,715
6	261X0416-6	AUX PASS FRAMES	2	26.7	\$527,700	\$178,783	\$252,920	\$959,403
4	261X0415-901,1	PANEL BOND ASSY.	1	66.7	\$4,680,948	\$621,000	\$11,031,137	\$16,333,085
6	261X0414-1,3	LWR SILL OUT CHRDR	2	5.3	\$140,000	\$99,216	\$113,732	\$352,948
6	261X0414-2	LWR SILL OUT CHRDR	1	8.3	\$170,000	\$161,352	\$48,244	\$379,596
6	261X0414-4,6	UPP SILL OUT CHRDR	2	5.3	\$140,000	\$107,265	\$113,732	\$360,997
6	261X0414-5	UPP SILL OUT CHRDR	1	8.4	\$172,000	\$163,296	\$48,244	\$383,540
6	261X0414-7,8	LWR SILL INR CHRDR	2	3.1	\$92,000	\$84,226	\$57,885	\$234,111
6	261X0414-9,10	EDGE FRAME	1	76.4	\$1,194,000	\$536,328	\$401,774	\$2,132,102
6	261X0419-11 TO 26	INTERCOSTALS	0	0.0	\$0	\$0	\$0	\$0
6	261X0419-27,28	SHEAR TIES	2	0.4	\$9,000	\$5,654	\$17,119	\$31,773
6	261X0414-35 TO 42	BACKUP FITTING	16	5.7	\$72,000	\$82,696	\$39,271	\$193,967
6	261X0414-47	ANGLE	2	0.1	\$1,875	\$1,200	\$16,667	\$19,742
6	261X0413-3,4,5,21	FRAME SPLICE	8	1.7	\$130,000	\$16,708	\$201,791	\$348,499
6	261X0413-6	SILL BRACKET	1	0.1	\$3,000	\$1,163	\$15,989	\$20,152
6	261X0413-7	SILL BRACKET	1	0.2	\$5,000	\$2,697	\$19,379	\$27,076
6	261X0413-8	SILL BRACKET	1	0.1	\$3,000	\$1,488	\$12,372	\$16,860
6	261X0413-9	SILL BRACKET	1	0.1	\$3,000	\$1,748	\$13,503	\$18,251
6	261X0413-20	BRACKET	2	0.0	\$2,000	\$949	\$14,407	\$17,356
6	261X0413-10,11	AL FITTING	2	1.1	\$210,000	\$86,652	\$36,225	\$332,877
6	261X0413-12,13	AL FITTING	2	1.3	\$195,000	\$28,188	\$30,705	\$253,893
6	261X0413-14,15	AL FITTING	2	1.1	\$570,000	\$85,180	\$78,919	\$734,099
6	261X0413-16,17	AL FITTING	2	1.0	\$495,000	\$52,992	\$76,504	\$624,496
6	261X0413-18,19	AL HINGE FITTING	2	4.6	\$1,260,000	\$263,736	\$197,168	\$1,720,904
5	417-1A	2D BRAID WIN. FRM.	14	28.0	\$3,025,500	\$598,500	\$295,000	\$3,919,000
5	261X0413-9061	F/G ISOLATION FAB	0	0.0	\$0	\$0	\$0	\$0
5	261X0413-906	F/G ISOLATION BOND	0	0.0	\$0	\$0	\$0	\$0
4	261X0413-905	WINDOW FRM INSTL.	0	0.0	\$0	\$0	\$0	\$0
10	261X0413-904	INTERCOSTAL ASSY.	0	0.0	\$0	\$0	\$0	\$0
10	261X0413-903	LWR SILL ASSY.	1	0.4	\$67,442	\$24,123	\$36,686	\$128,251
10	261X0413-902	UPR SILL ASSY.	1	0.4	\$68,250	\$25,768	\$36,686	\$130,704
10	261X0413-901	DOOR STRCT ASSY.	1	1.0	\$275,742	\$61,760	\$486,753	\$824,255
10	261X0413-1	DOOR INSTL.	1	13.0	\$799,978	\$861,876	\$2,106,527	\$3,768,381
7	261X0422-9,10	PASS. FLOOR BEAM	10	273.1	\$208,352	\$1,460,403	\$26,387	\$1,695,141
7	261X0412-11,12	STANCHION	10	65.9	\$175,500	\$644,038	\$67,969	\$887,507
8	261X0412-14	AL CARGO FB SPLICE	10	6.9	\$1,425,000	\$659,919	\$73,830	\$2,158,749
8	261X0412-4,5,6	SPLICE PLATE	2	10.7	\$105,000	\$177,000	\$888,492	\$1,170,492
8	261X0412-7	FRAME SPLICE	10	5.6	\$175,200	\$47,175	\$189,561	\$411,935
8	261X0412-8	FRAME SPL. FILLER	10	0.5	\$14,000	\$8,579	\$300,394	\$322,972
8	261X0412-2,16	END FRAME	1	15.5	\$313,800	\$100,440	\$752,551	\$1,166,791
8	261X0412-3	SHEAR TIES	2	1.6	\$118,000	\$17,107	\$323,463	\$458,570
10	261X0412-908	LOWER LOBE ASSY.	0	0.0	\$0	\$0	\$0	\$0

SIDE PANEL COST & WEIGHT BREAKDOWN
FAMILY D (SANDWICH) - LOCAL OPTIMIZATION POTENTIAL
(Weight in lbs per shipset, Cost in \$ per 300 shipsets)

C	PART #	DESCRIPTION	QTY	WEIGHT	RECURRING LABOR	RECURRING MATERIAL	NON RECURRING	TOTAL
10	261X0412-907	PASS. FB INSTL.	1	1.4	\$107,536	\$82,165	\$6,767	\$196,467
10	261X0412-906	PASS. STAN. INSTL.	1	1.6	\$155,033	\$93,721	\$13,533	\$262,287
10	261X0412-905	LONG. LAP SPLICE	0	0.0	\$0	\$0	\$0	\$0
10	261X0412-904	FRAME SPLICE	1	2.1	\$201,275	\$126,411	\$9,022	\$336,708
10	261X0412-903	CARGO FB INSTL.	1	4.9	\$335,857	\$276,076	\$0	\$611,932
10	261X0412-902	OVRWNG LNG INSTL	1	1.4	\$58,921	\$109,163	\$0	\$168,084
10	261X0412-901	46/47 SEC. JOIN	1	7.1	\$572,716	\$402,441	\$485,578	\$1,460,735
10	261X0412-1	46/45 SEC. JOIN	1	6.9	\$572,716	\$387,391	\$485,578	\$1,445,685
TOTAL				1831.4	\$30,708,496	\$24,100,903	\$21,387,291	\$76,196,690

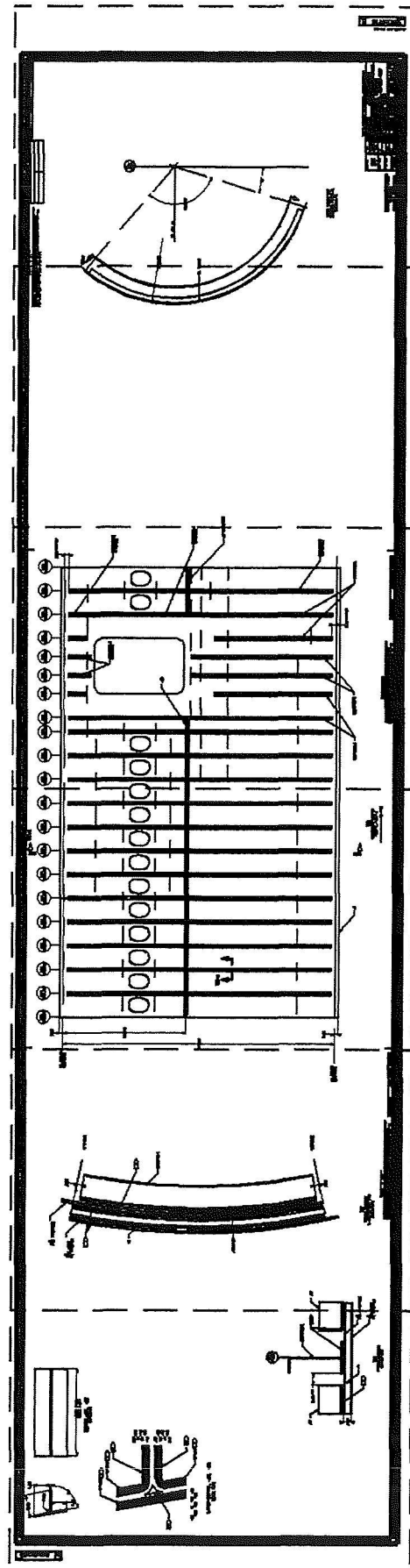
SUMMARY

C	DESCRIPTION	WEIGHT	RECURRING LABOR	RECURRING MATERIAL	NON RECURRING	TOTAL
1	Skin	821.8	\$1,685,309	\$9,508,595	\$73,711	\$11,267,615
2	Stringers/Core	145.6	\$6,419,700	\$5,151,430	\$993,861	\$12,564,991
3	Frames	188.1	\$3,033,150	\$590,809	\$735,756	\$4,359,715
5	Window Belt	38.3	\$3,744,500	\$702,000	\$456,500	\$4,903,000
4	Skin Panel Ass'y	66.7	\$4,680,948	\$621,000	\$11,031,137	\$16,333,085
6	Door Reinforce. Fab	151.0	\$5,394,575	\$1,961,517	\$1,806,547	\$9,162,639
7	Pass. Floor Details	339.0	\$383,852	\$2,104,440	\$94,356	\$2,582,647
8	Splice Details Fab	40.7	\$2,151,000	\$1,010,219	\$2,528,291	\$5,689,510
10	Assembly	40.2	\$3,215,463	\$2,450,894	\$3,667,131	\$9,333,488
TOTAL		1831.4	\$30,708,496	\$24,100,903	\$21,387,291	\$76,196,690

APPENDIX D

FAMILY D DESIGN DRAWINGS

D-1



D-2

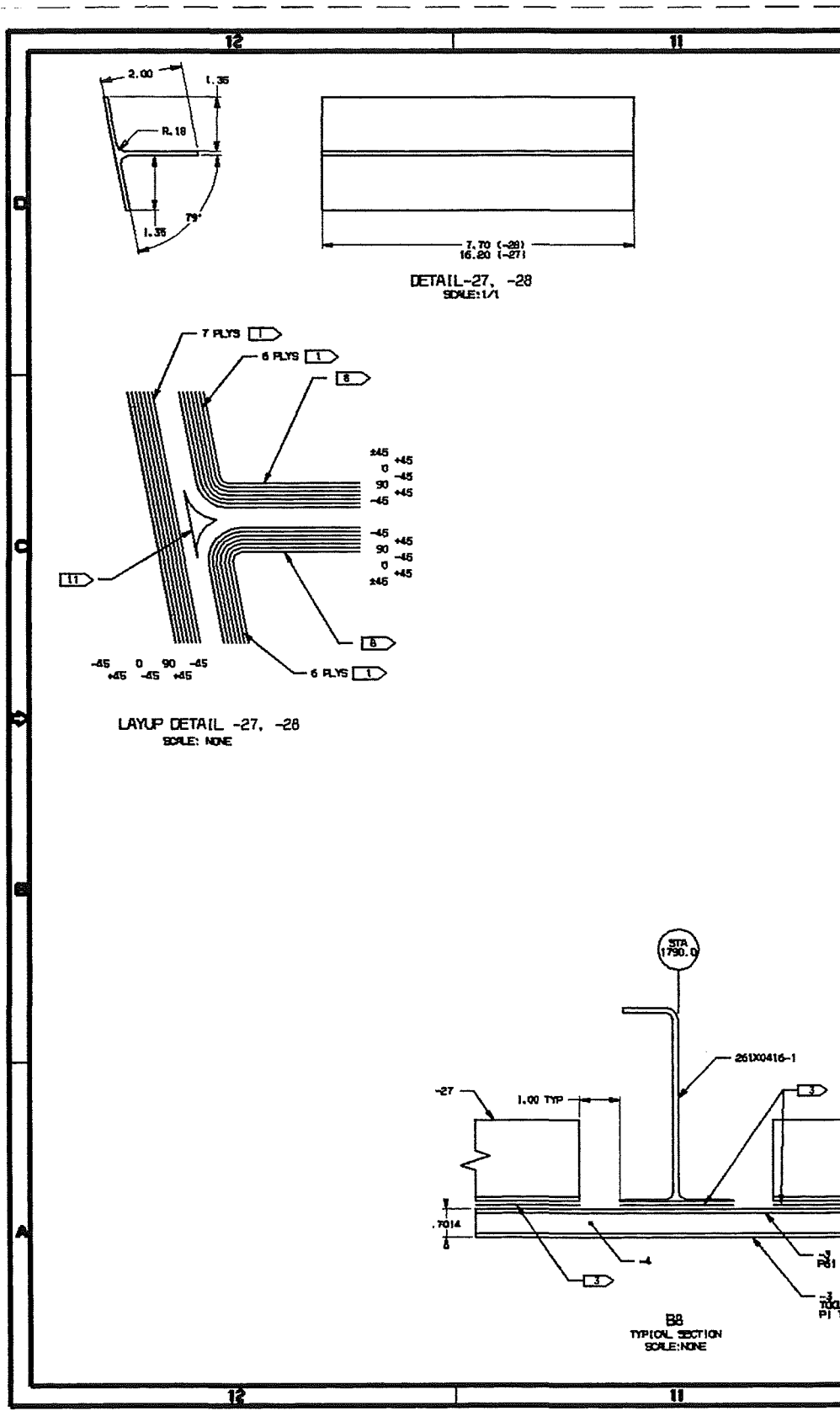
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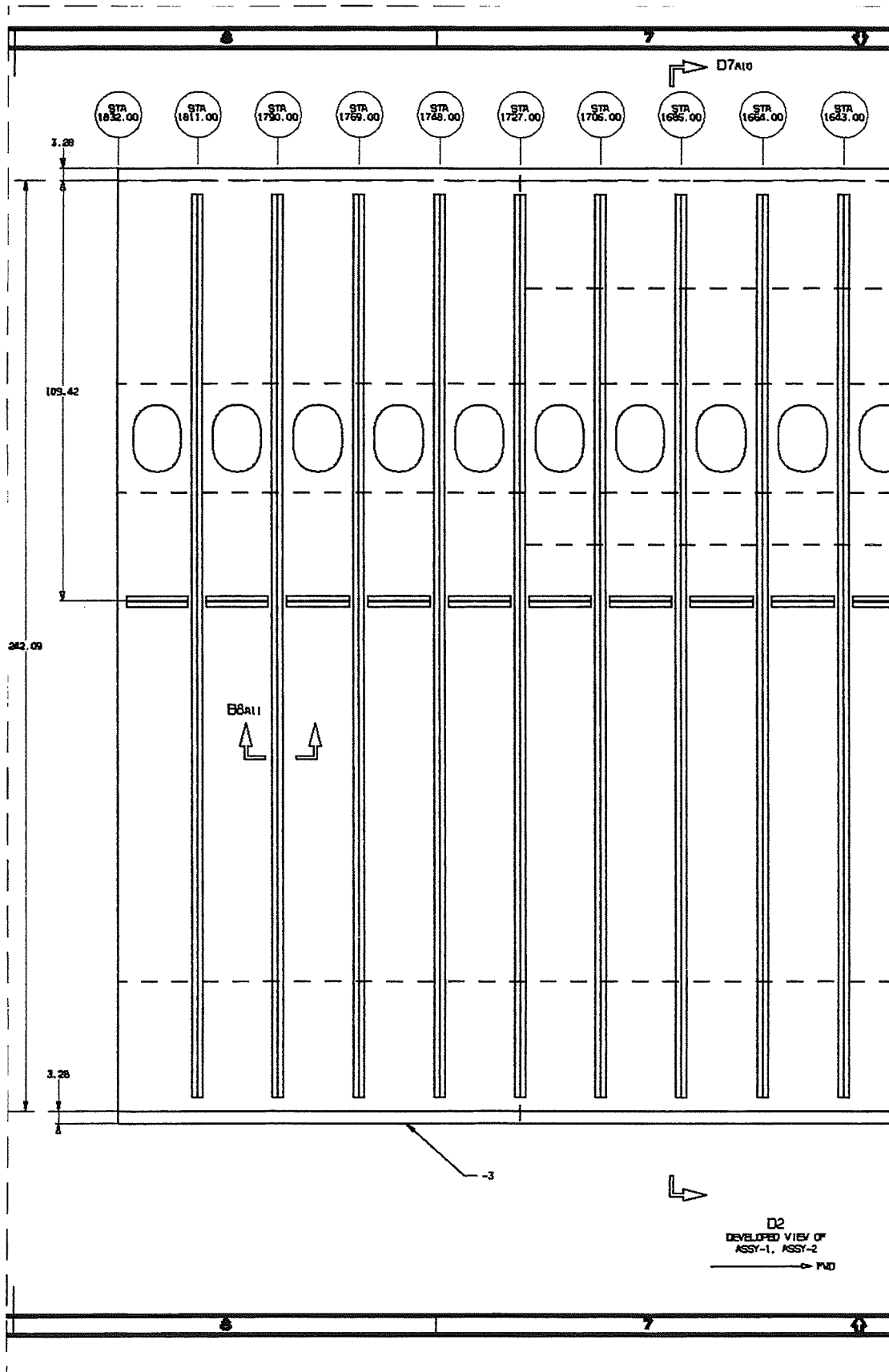
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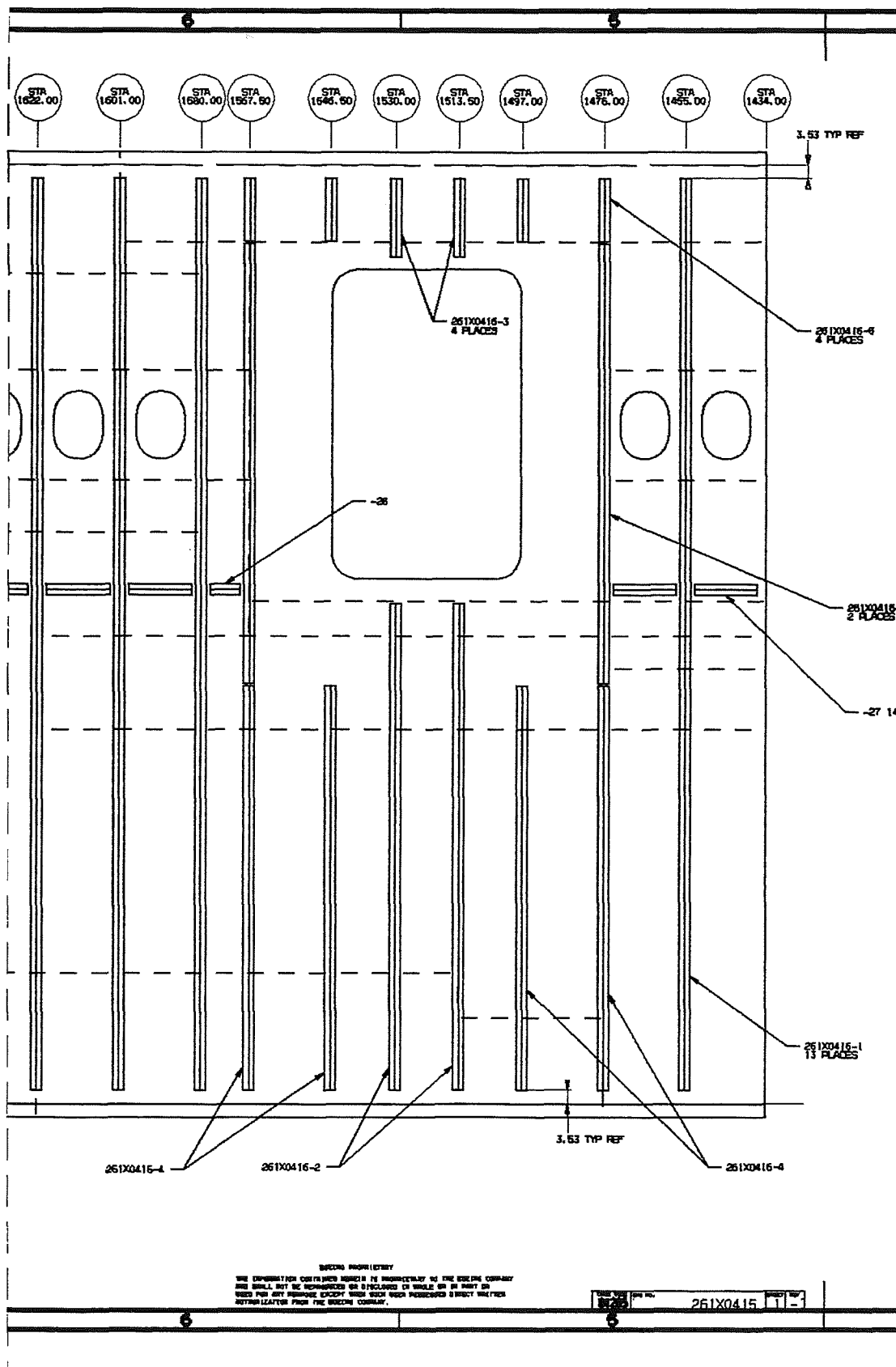
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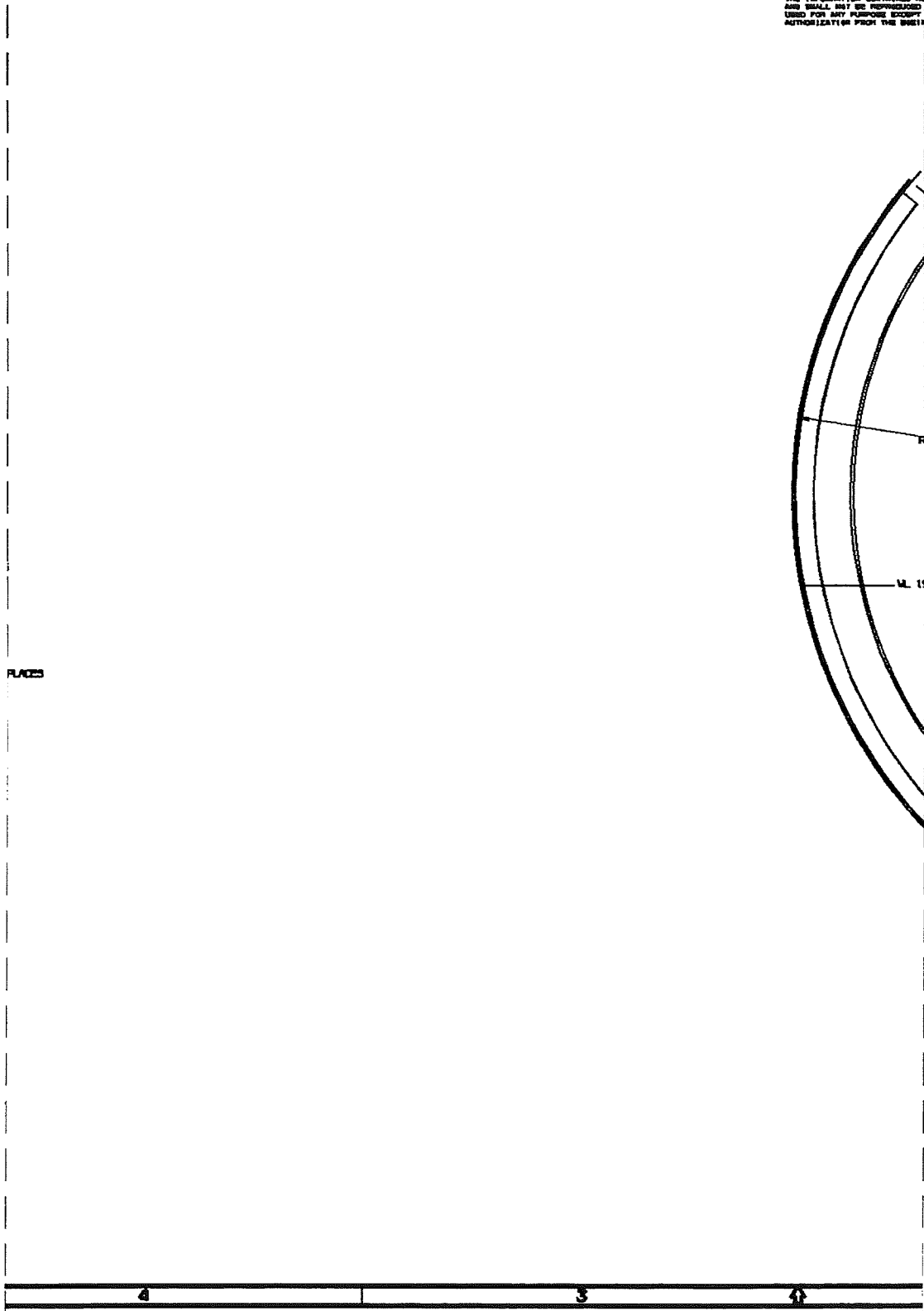
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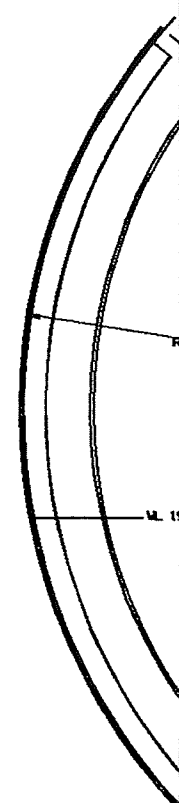
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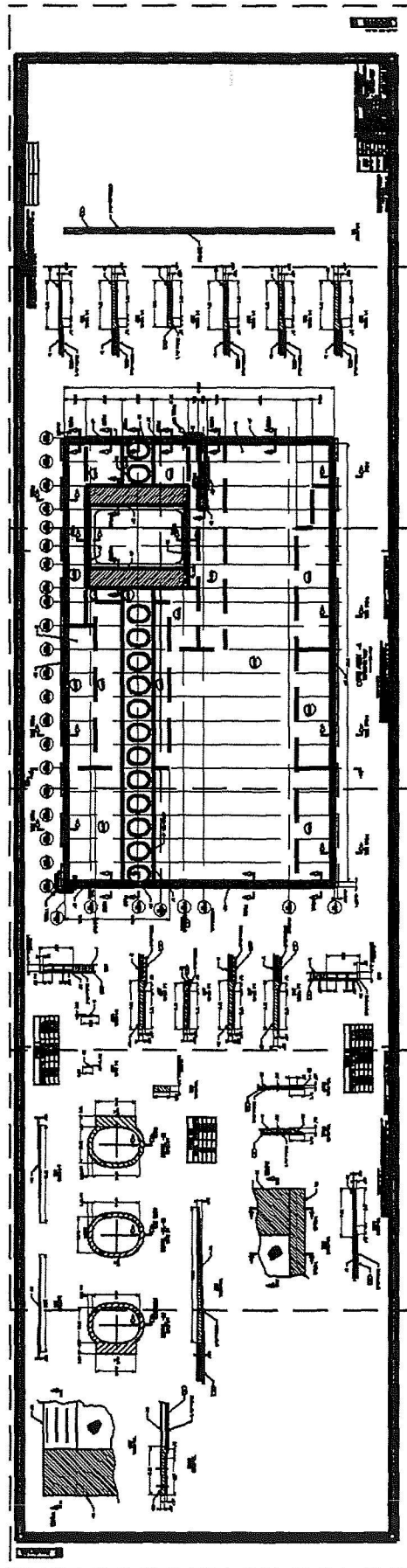


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<p>THE INFORMATION CONTAINED HEREIN IS UNCLASSIFIED EXCEPT WHERE SHOWN OTHERWISE BY THE MARKING.</p>					
					
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D-10

D-11

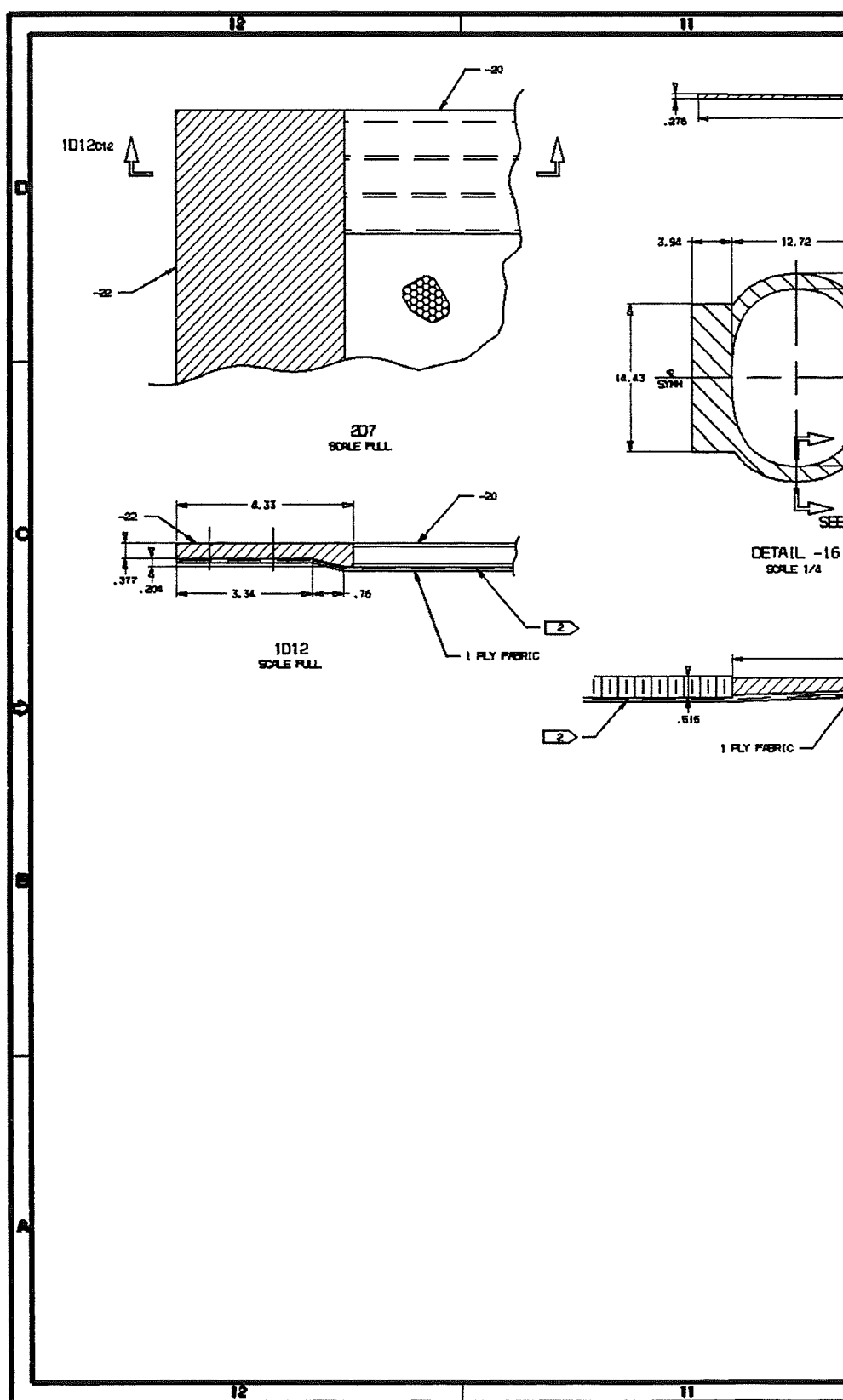
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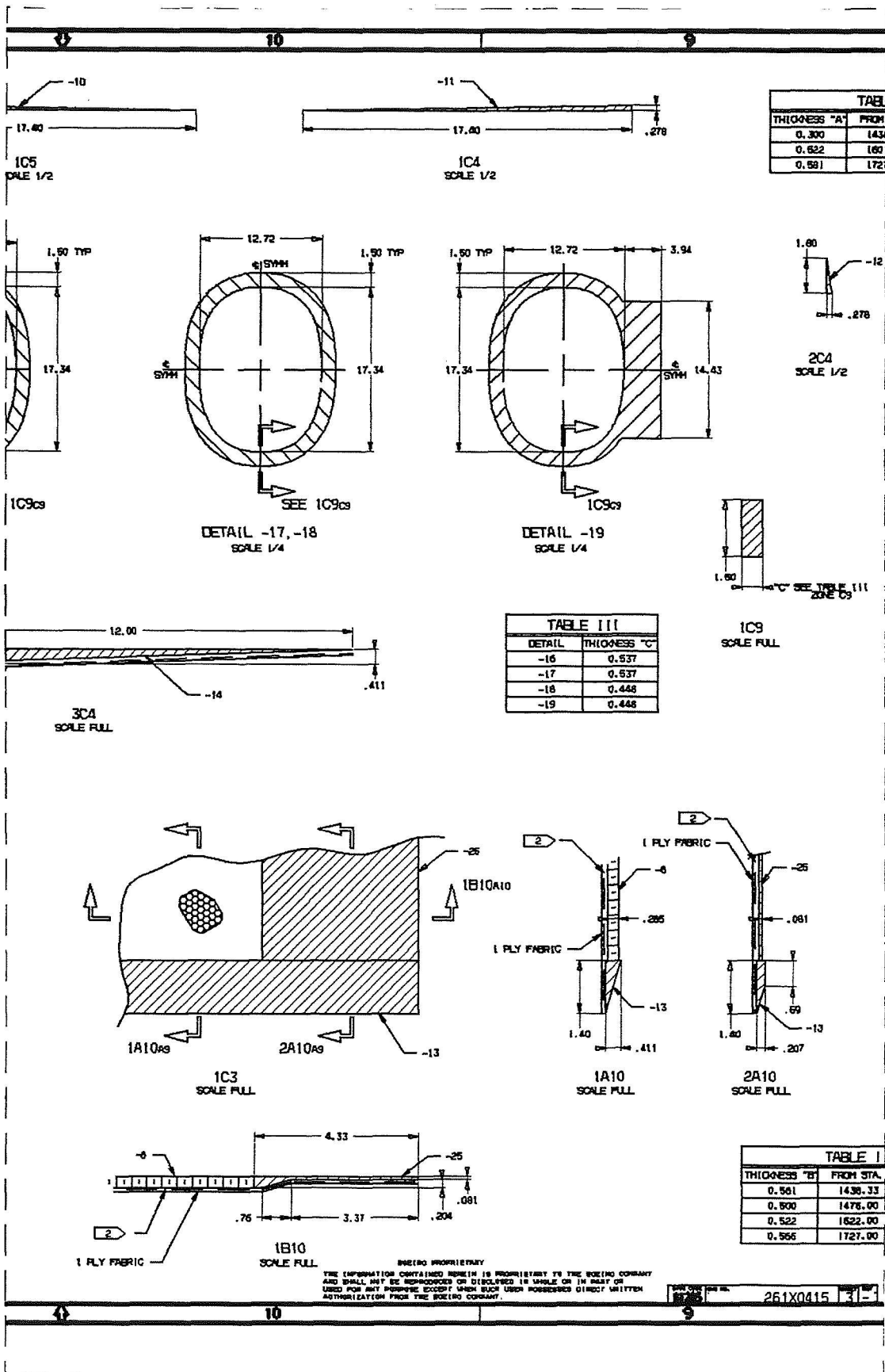
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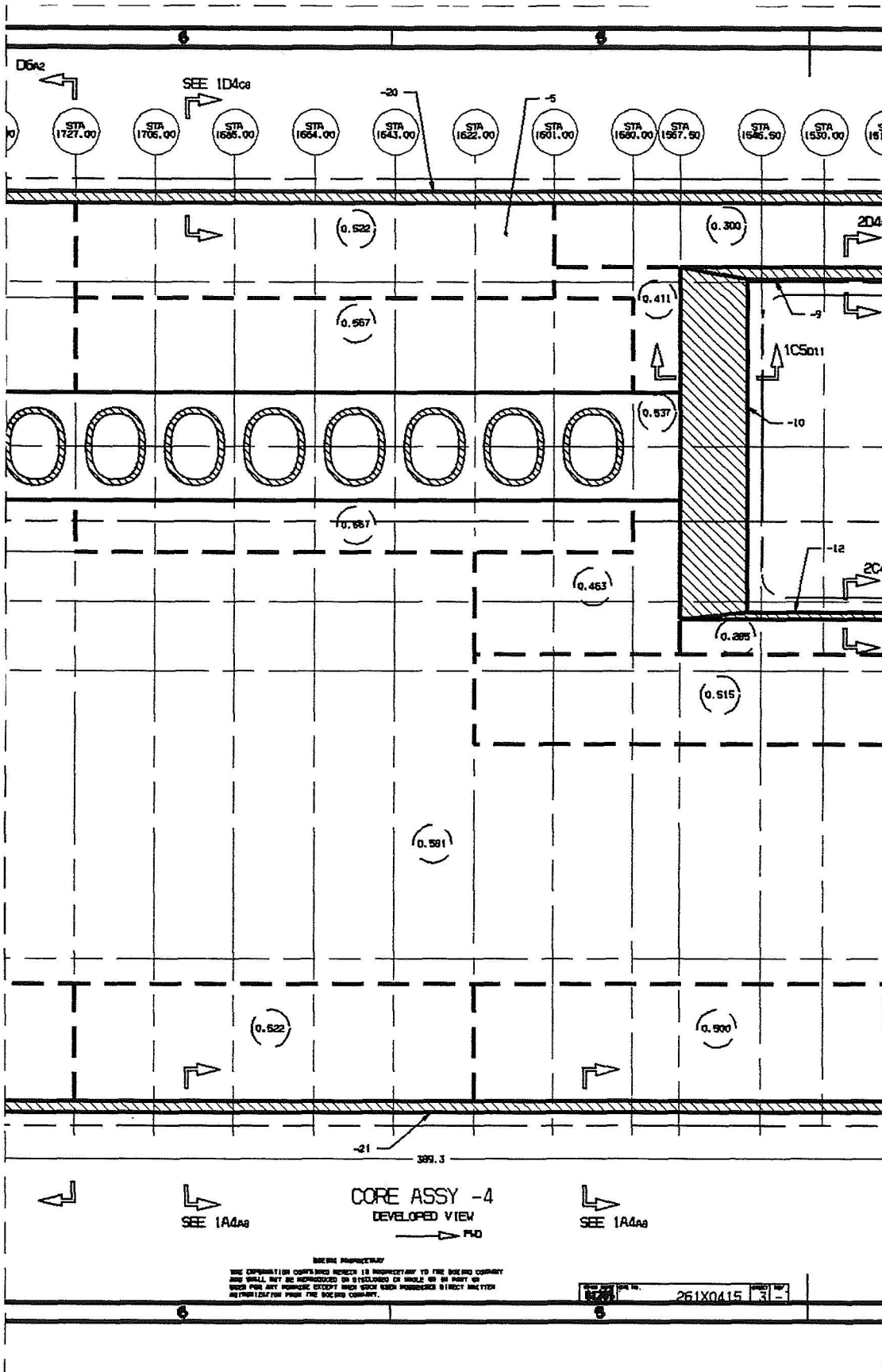
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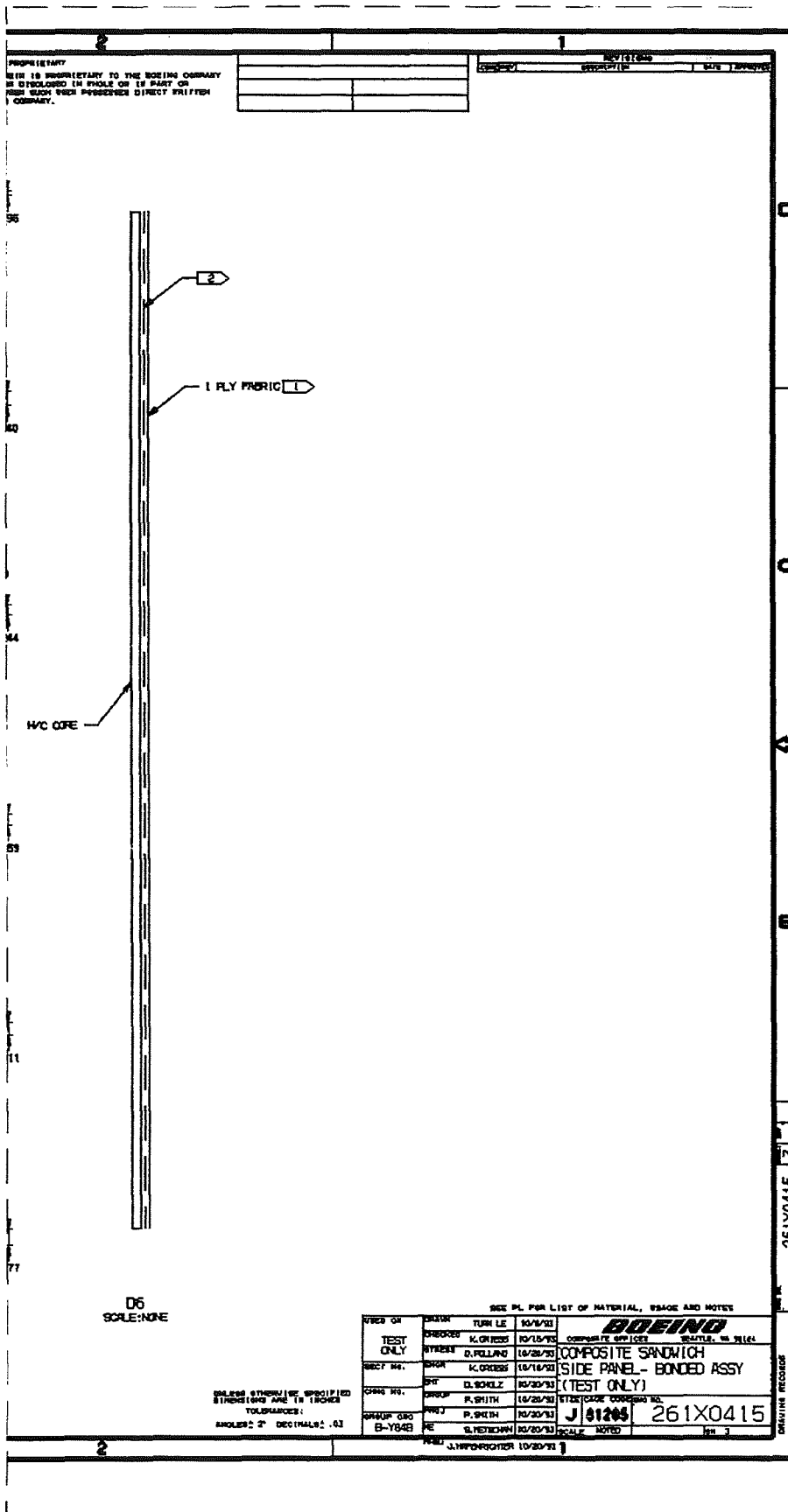


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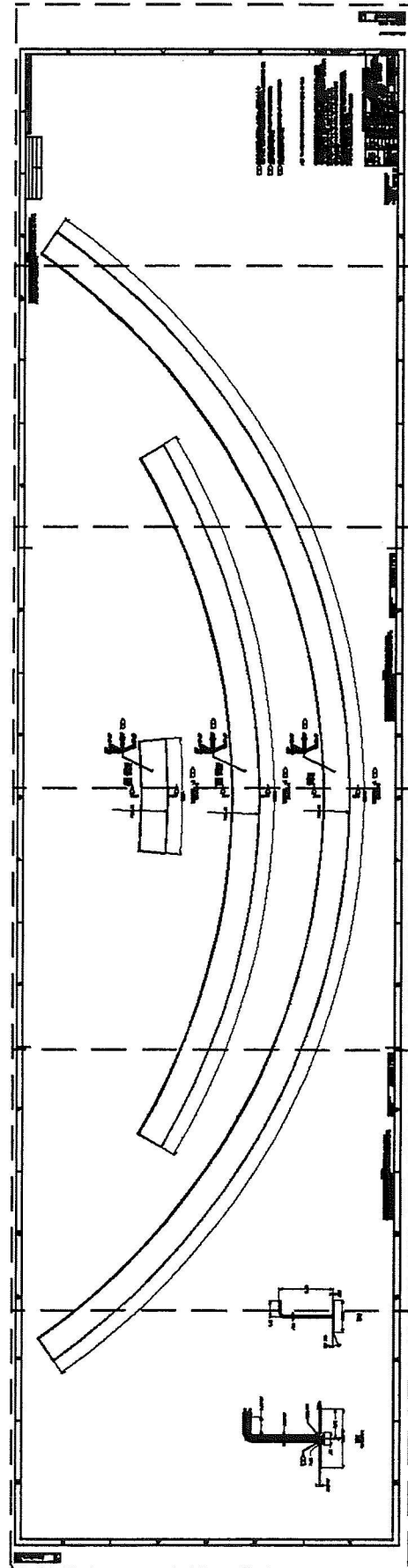






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D-17

D-18

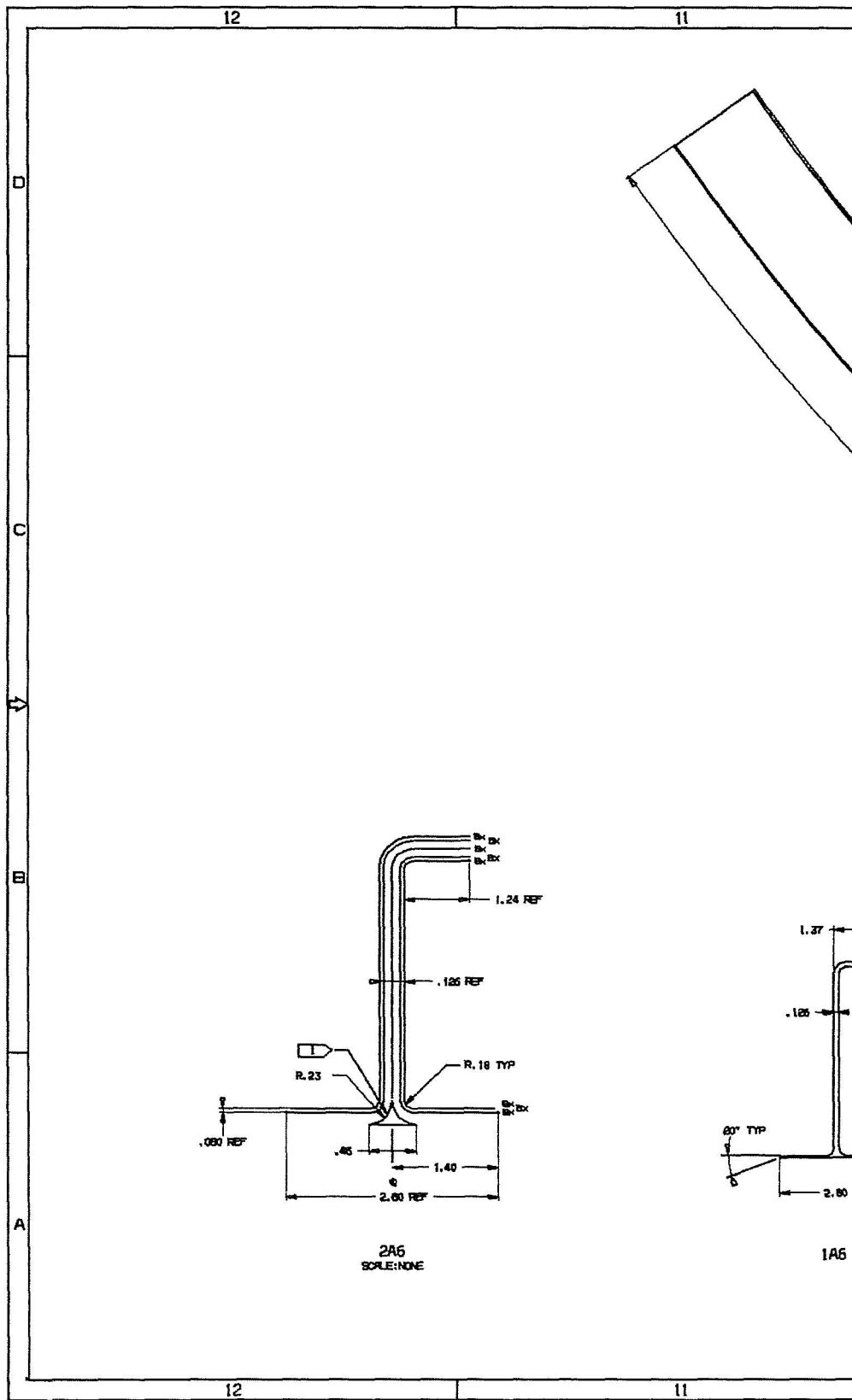
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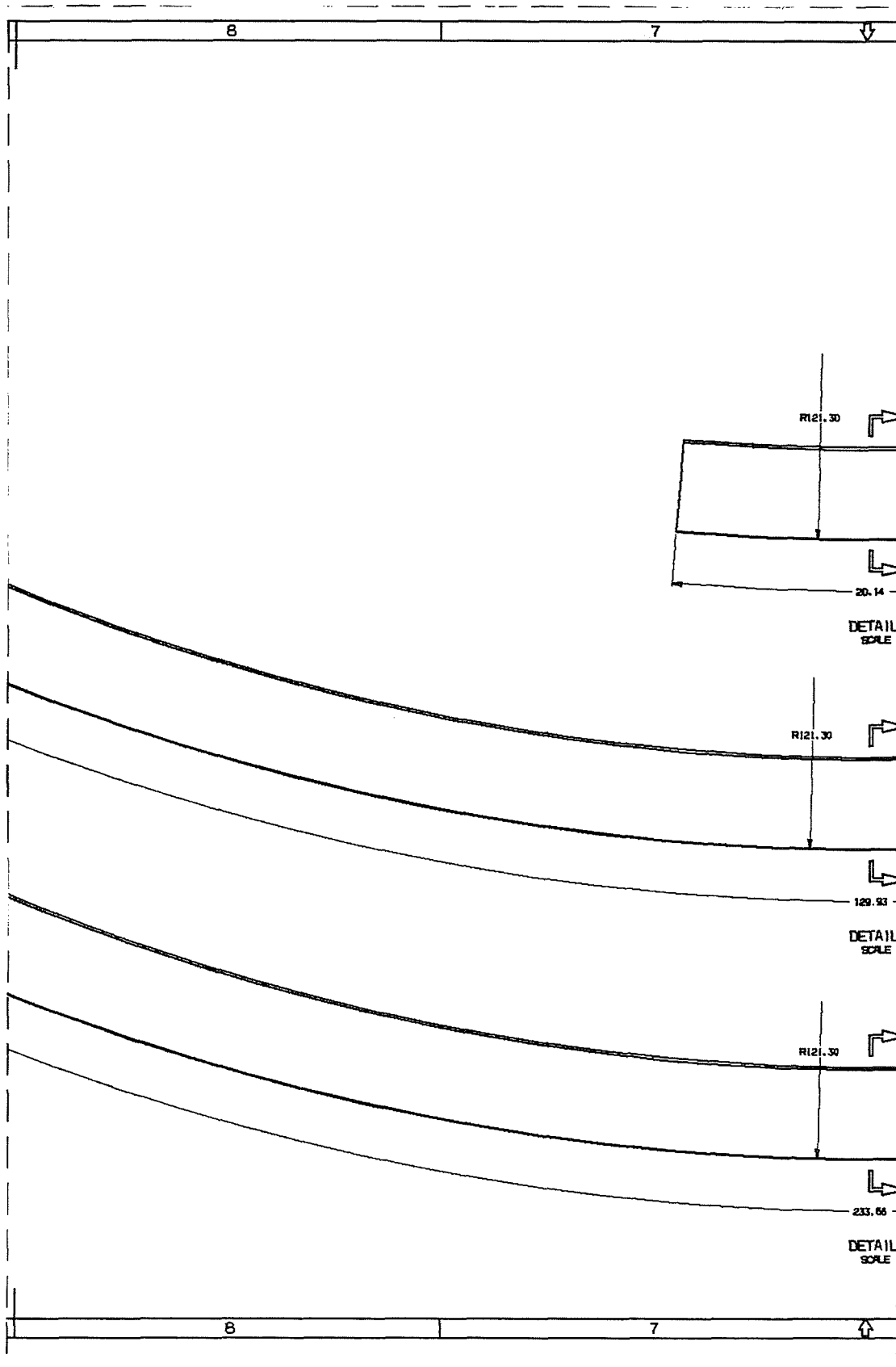
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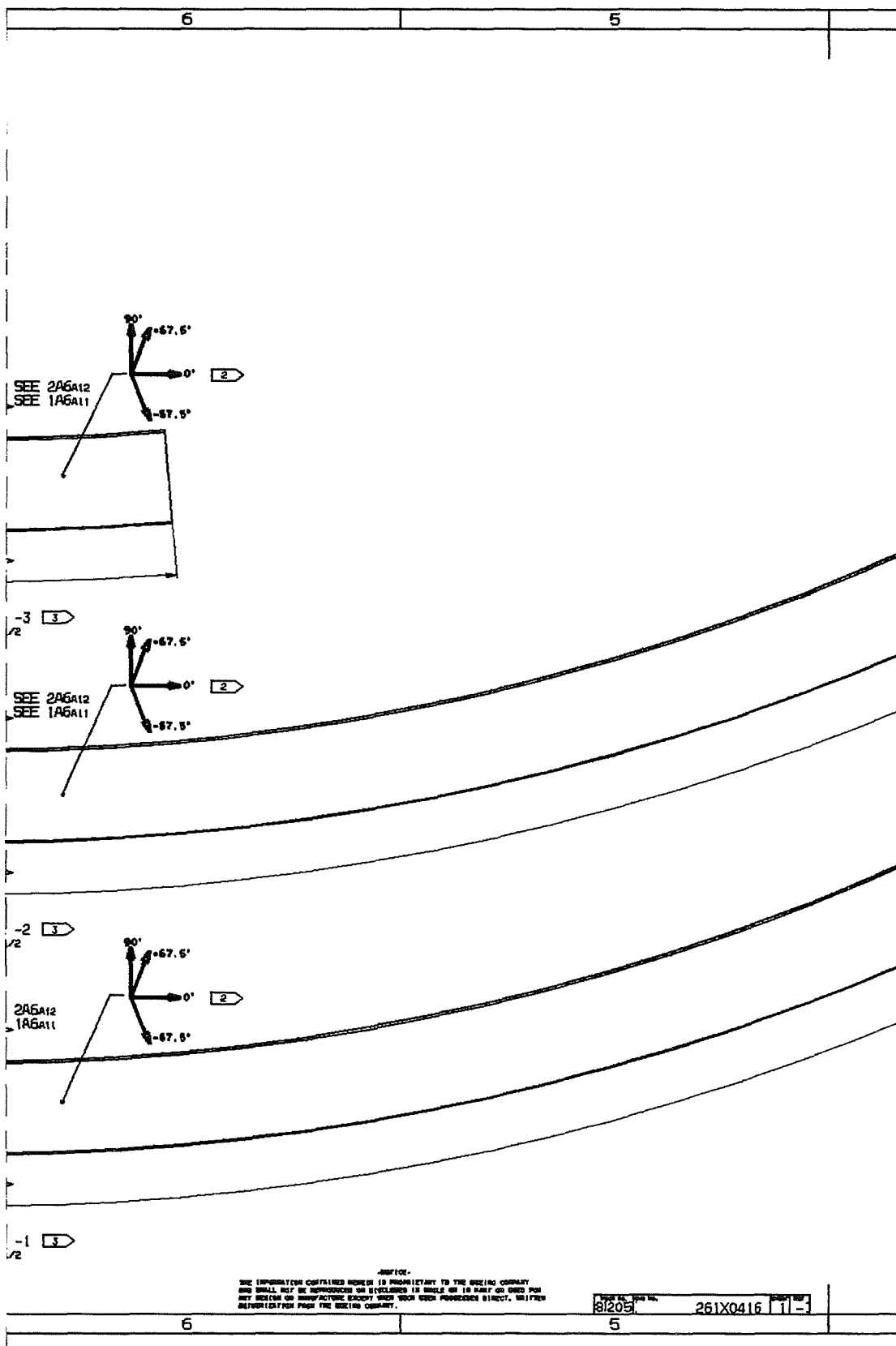
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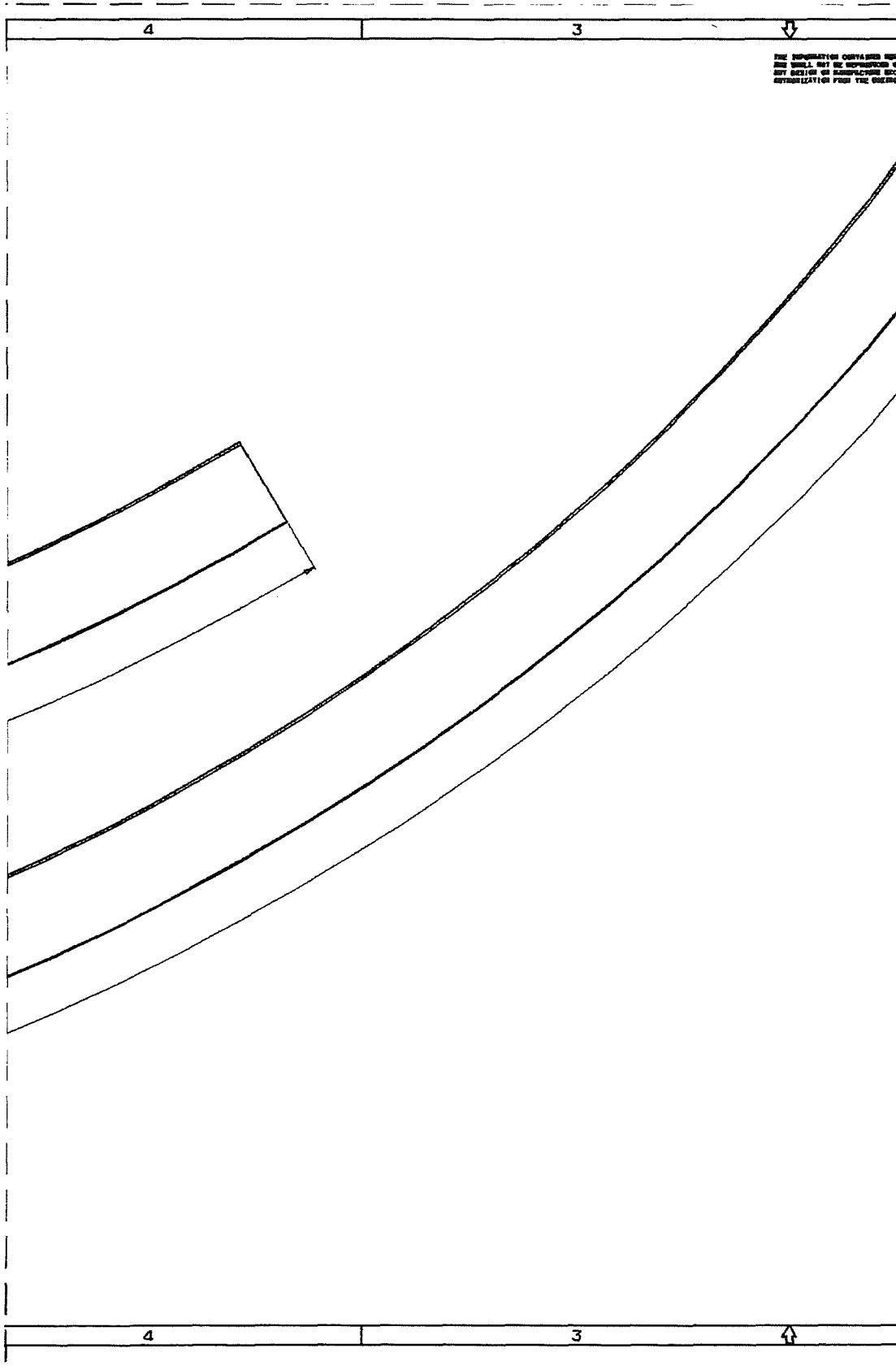
D-22

1 2140X125









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REVISIONS	DATE								

3 EPOXY RESIN (TYPE 1, CLASS 1, GRADE 12 & 6)
 EPOXY RESIN COMPATIBLE WITH RTH PROCESSING
 EX: BELL 1695 WITH EPOX CURING AGENT V AND ACCELERATOR 637.

2 FIBER ORIENTATION IS TANGENT TO THE CIRCUMFERENTIAL
 CURVATURE OF THE FRAME.

1 FILLER OF EPOXY RESIN ADHESIVE FORMED WITH A MOULDED EXPLORER
 MOULDED CONFIGURATION TEE.

-12K TOY GROUPS ARE MADE BY COMBINING 12K & 6K TOYS.

BRIDGED REINFORCEMENT NOTATIONS: BY ARE PILES OF TRIAXIAL BRAID
 WITH FIBER ORIENTATIONS -45, 0, -45 & TYP 67, 67, 144 CARRIER
 BRAIDED ONE 6K TOY/CARRIER WITH 72 3K TOY GROUPS IN-LAID
 IN 0° DIRECTION. TYPICAL PER PLY THICKNESS FOR VT- SEE 0.024".

-TOLERANCE ON PLY EDGE ± .10 EXCEPT AS NOTED.

-SURFACE FINISH ALL MACHINED SURFACES 125/RAK.

-TRIM AND MACHINE OR/EP PARTS TO THE REQUIREMENTS
 OF BAC 5670.

-BREAK ALL SHARP EDGES AND CORNERS ON OR/EP DETAILS TO
 A RADIUS OR CHAMFER OF APPROX .01. VISUAL INSPECT ONLY.

-PART MARK PER BAC5507. CODE N.

-FABRICATE PER BAC 5670. (WHERE APPLICABLE)

2

SEE PL FOR LIST OF MATERIAL, USAGE AND NOTES

TEST ONLY	NAME	DATE	DATE	DATE	DATE	DATE	DATE
	AL. GREGG	4/8/91					
	D. HENCH	10/19/92					
	D. POLLARD	10/19/92					
	K. GREGG	10/19/92					
	D. BOOLE	10/19/92					
	P. SMITH	10/19/92					
	P. SMITH	10/19/92					
	K. VILLON	10/19/92					

261X0416

**COMPOSITE SANDWICH SIDE
 PANEL - FRAME DETAILS
 (TEST ONLY)**

J 81205

261X0416

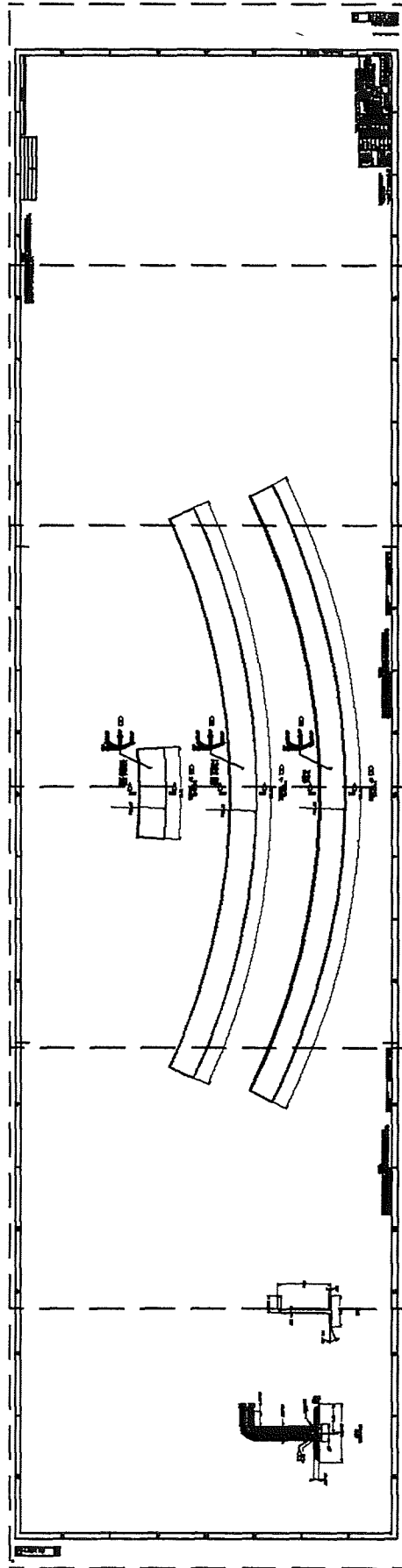
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D-29

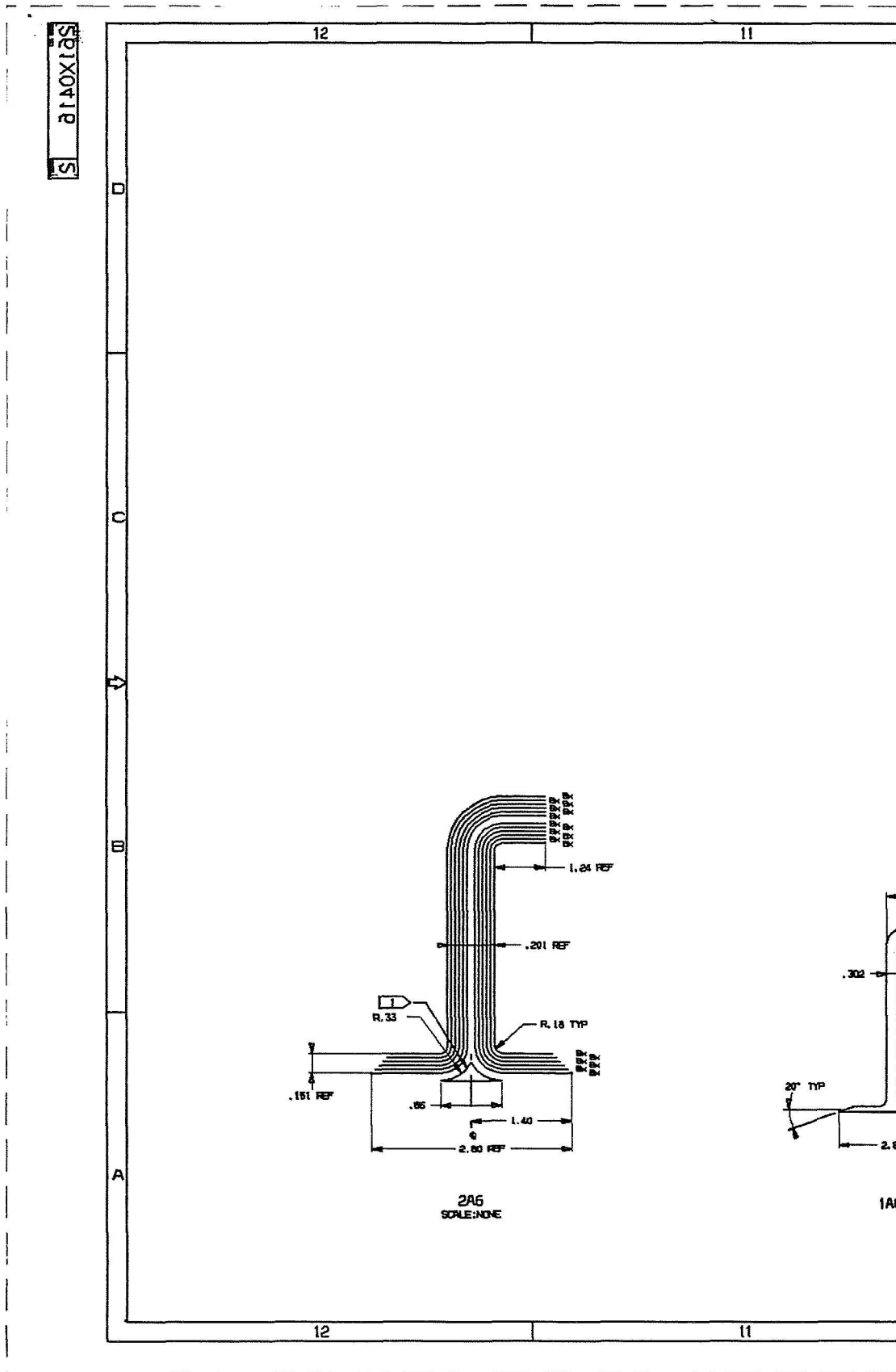
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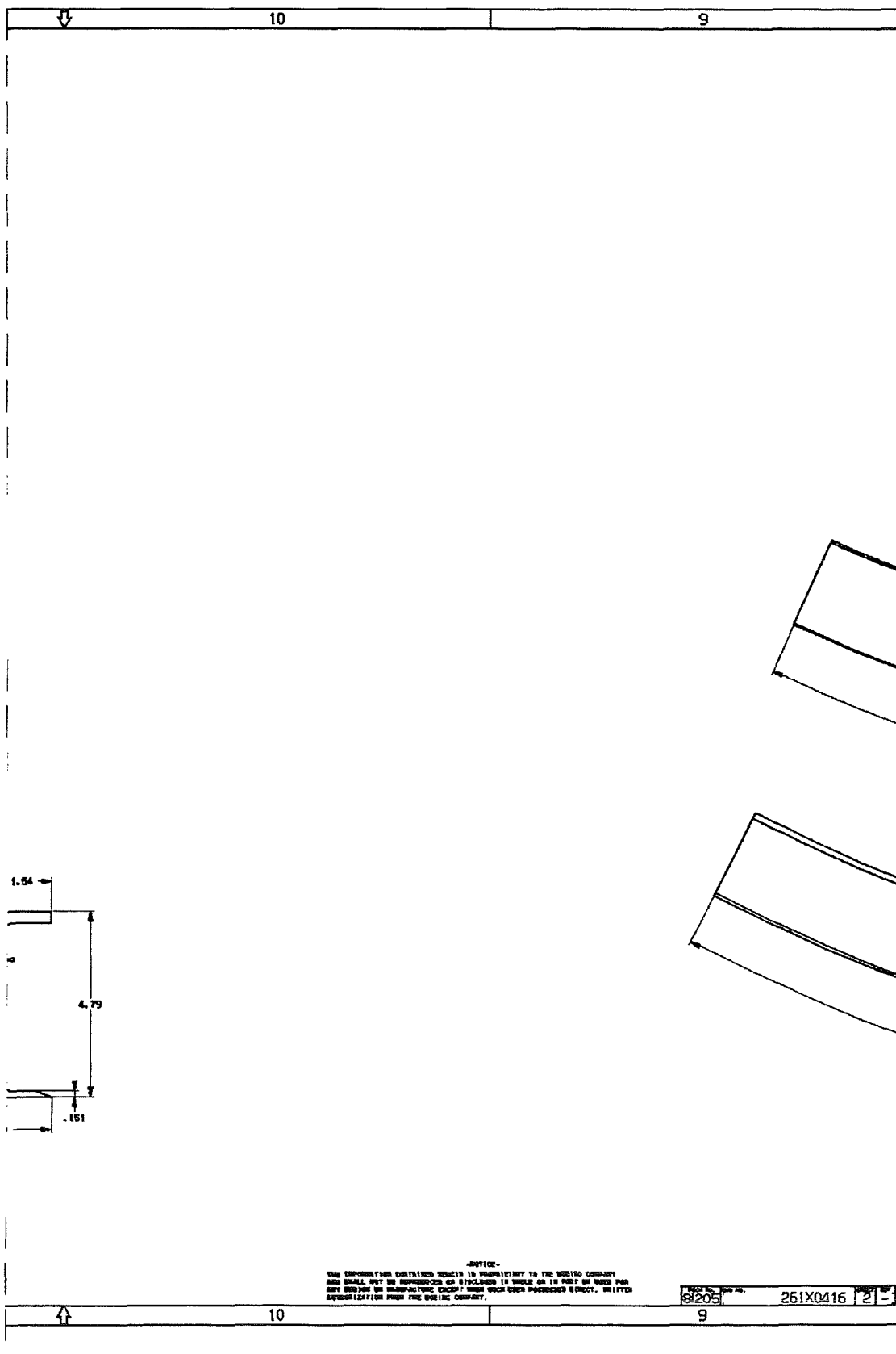
D-27

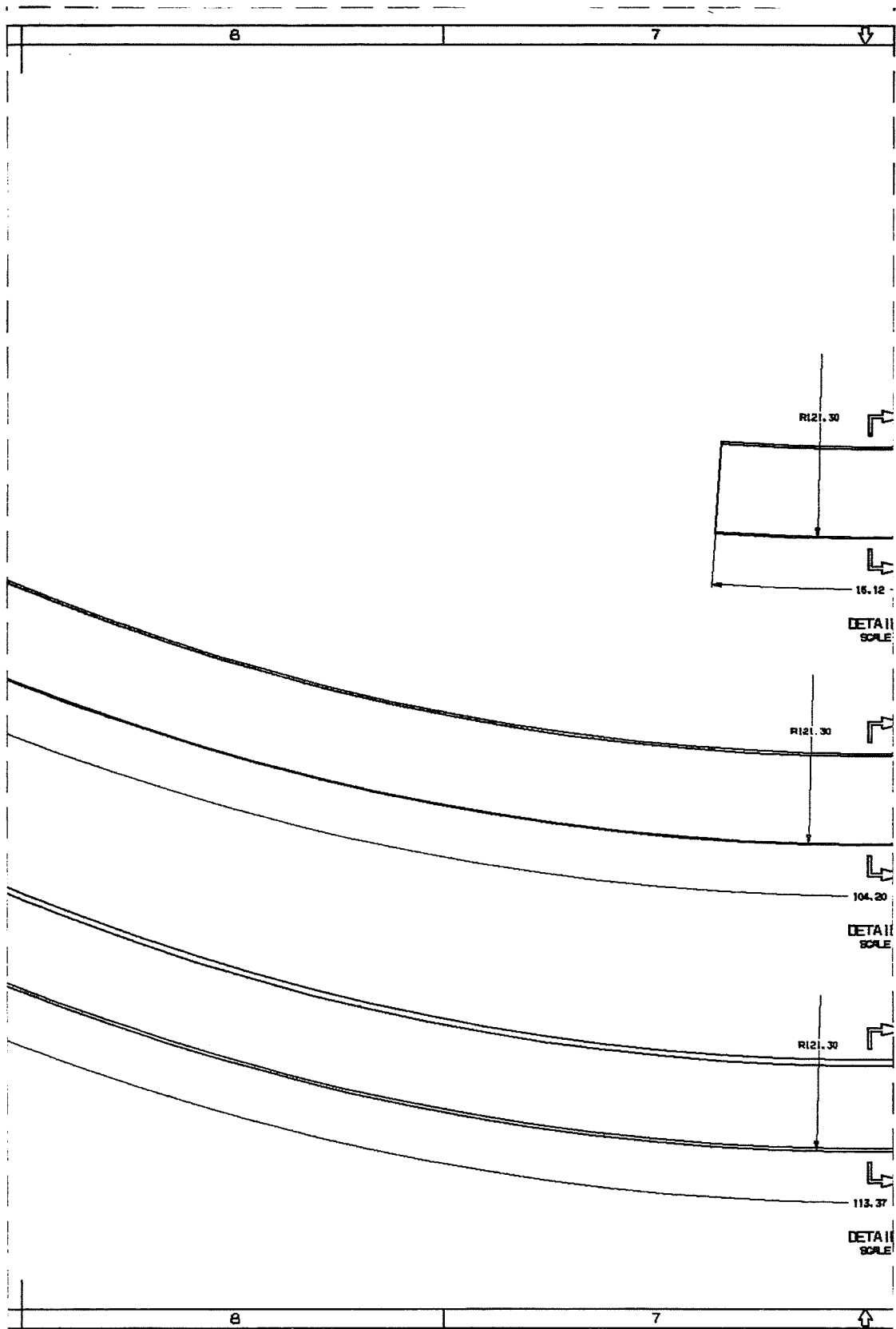
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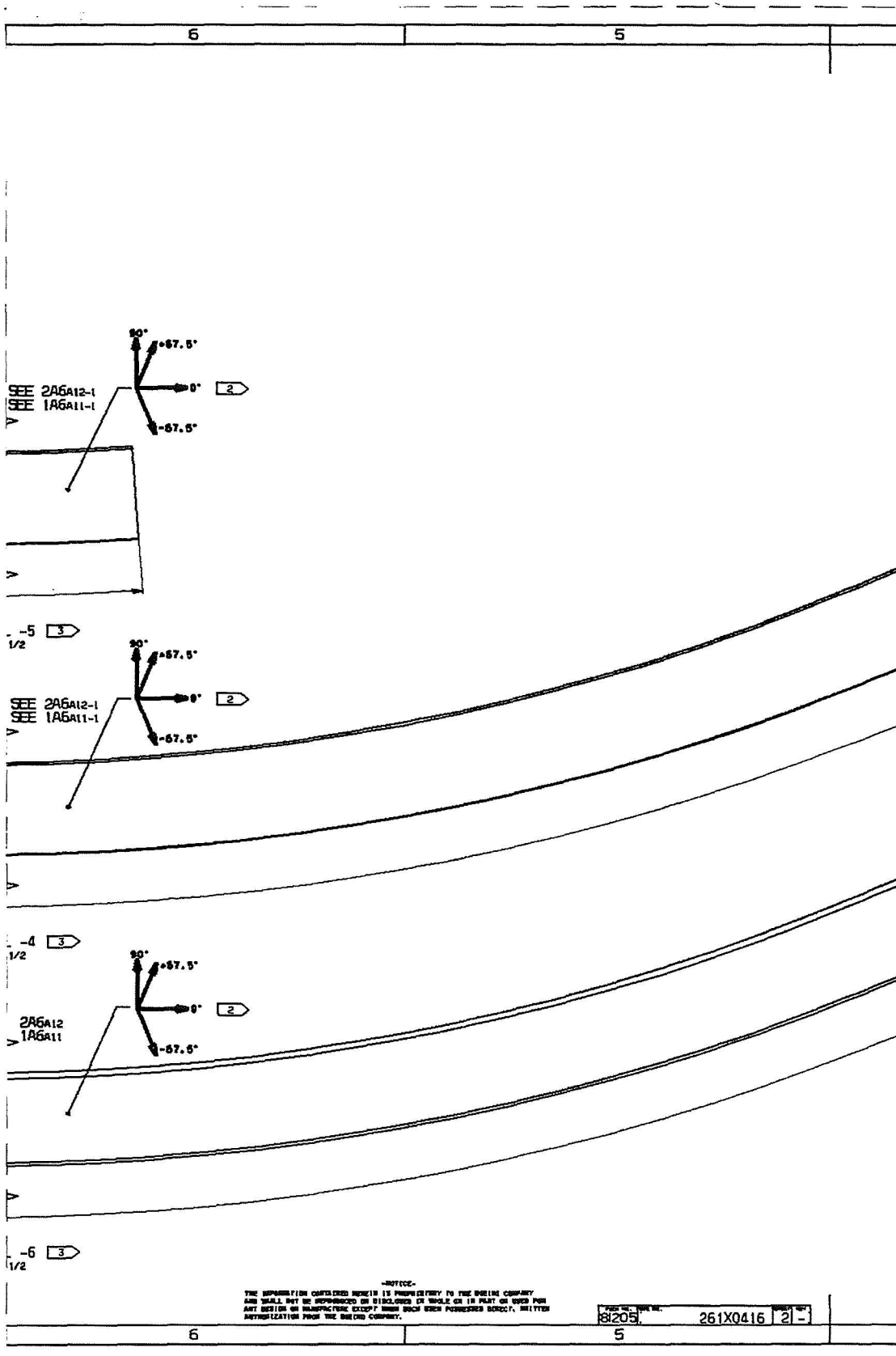
D-25

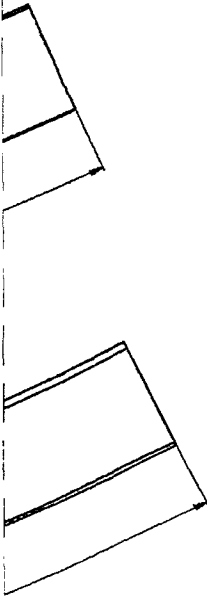
D-24



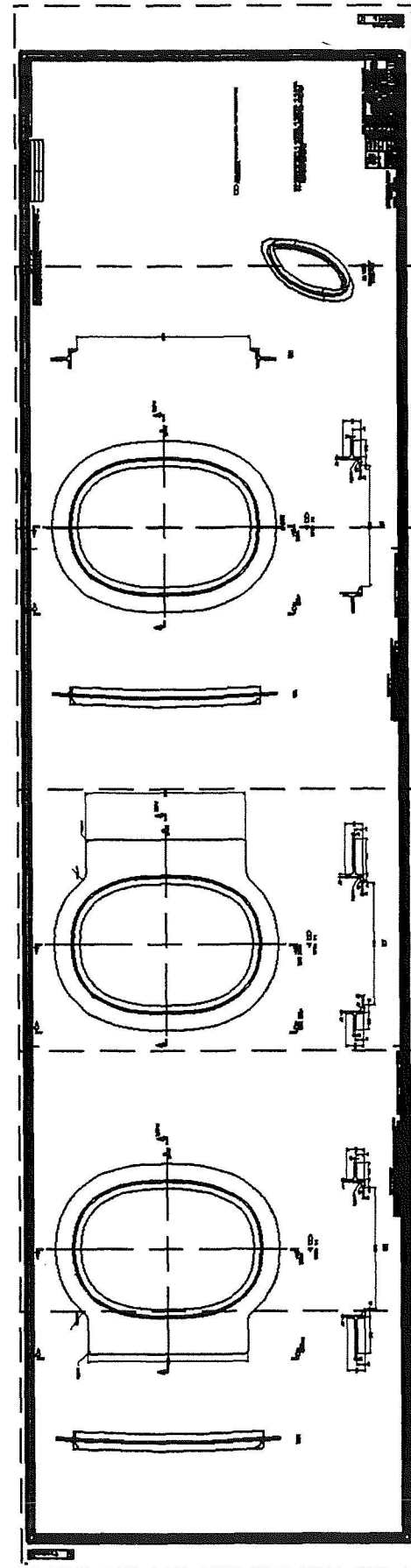






4	3	↓
<div style="position: absolute; top: 10px; right: 10px; font-size: 8px;"> THE INFORMATION CONTAINED HEREIN IS UNCLASSIFIED EXCEPT WHERE SHOWN OTHERWISE BY THE SOURCE </div> <div style="position: absolute; bottom: 10px; left: 10px;">  </div>		
4	3	↑

2	1																				
<p>NOTICE- THIS DRAWING IS THE PROPERTY OF THE BOEING COMPANY IT IS TO BE USED ONLY FOR THE PURPOSES SPECIFIED HEREON IT IS NOT TO BE REPRODUCED OR TRANSMITTED IN ANY FORM OR BY ANY MEANS WITHOUT PERMISSION IN WRITING FROM THE BOEING COMPANY.</p>	<table border="1" style="width: 100%; border-collapse: collapse;"> <tr> <td style="width: 50%;"> REVISIONS NO. DATE BY DESCRIPTION </td> <td style="width: 50%;"> DATE APPROVED </td> </tr> <tr> <td> </td> <td> </td> </tr> <tr> <td> </td> <td> </td> </tr> <tr> <td> </td> <td> </td> </tr> </table>	REVISIONS NO. DATE BY DESCRIPTION	DATE APPROVED																		
REVISIONS NO. DATE BY DESCRIPTION	DATE APPROVED																				
<div style="display: flex; justify-content: space-between; align-items: flex-end;"> <div style="width: 30%;"> <p>UNLESS OTHERWISE SPECIFIED DIMENSIONS ARE IN INCHES TOLERANCES: FINISHES: 2" DECIMALS: .03</p> </div> <div style="width: 65%;"> <table border="1" style="width: 100%; border-collapse: collapse;"> <tr> <td colspan="4" style="text-align: center;">SEE PL. FOR LIST OF MATERIAL, USAGE AND NOTES</td> </tr> <tr> <td style="width: 15%;"> TEST ONLY </td> <td style="width: 15%;"> DESIGNED BY K. GORDON </td> <td style="width: 15%;"> DATE 4/8/51 </td> <td style="width: 55%;"> BOEING COMPOSITE SANDWICH SIDE PANEL - FRAME DETAILS (TEST ONLY) </td> </tr> <tr> <td> CHECKED BY K. GORDON </td> <td> DATE 10/15/53 </td> <td> DRAWN BY K. GORDON </td> <td> DATE 10/15/53 </td> </tr> <tr> <td> INCHES 1/8" </td> <td> FEET 0 </td> <td> TOLERANCE .03 </td> <td> PARTIAL RECORDS </td> </tr> <tr> <td> PARTIAL RECORDS </td> <td> PARTIAL RECORDS </td> <td> PARTIAL RECORDS </td> <td> PARTIAL RECORDS </td> </tr> </table> </div> </div>		SEE PL. FOR LIST OF MATERIAL, USAGE AND NOTES				TEST ONLY	DESIGNED BY K. GORDON	DATE 4/8/51	BOEING COMPOSITE SANDWICH SIDE PANEL - FRAME DETAILS (TEST ONLY)	CHECKED BY K. GORDON	DATE 10/15/53	DRAWN BY K. GORDON	DATE 10/15/53	INCHES 1/8"	FEET 0	TOLERANCE .03	PARTIAL RECORDS	PARTIAL RECORDS	PARTIAL RECORDS	PARTIAL RECORDS	PARTIAL RECORDS
SEE PL. FOR LIST OF MATERIAL, USAGE AND NOTES																					
TEST ONLY	DESIGNED BY K. GORDON	DATE 4/8/51	BOEING COMPOSITE SANDWICH SIDE PANEL - FRAME DETAILS (TEST ONLY)																		
CHECKED BY K. GORDON	DATE 10/15/53	DRAWN BY K. GORDON	DATE 10/15/53																		
INCHES 1/8"	FEET 0	TOLERANCE .03	PARTIAL RECORDS																		
PARTIAL RECORDS	PARTIAL RECORDS	PARTIAL RECORDS	PARTIAL RECORDS																		
2	1																				



D-36

D-35

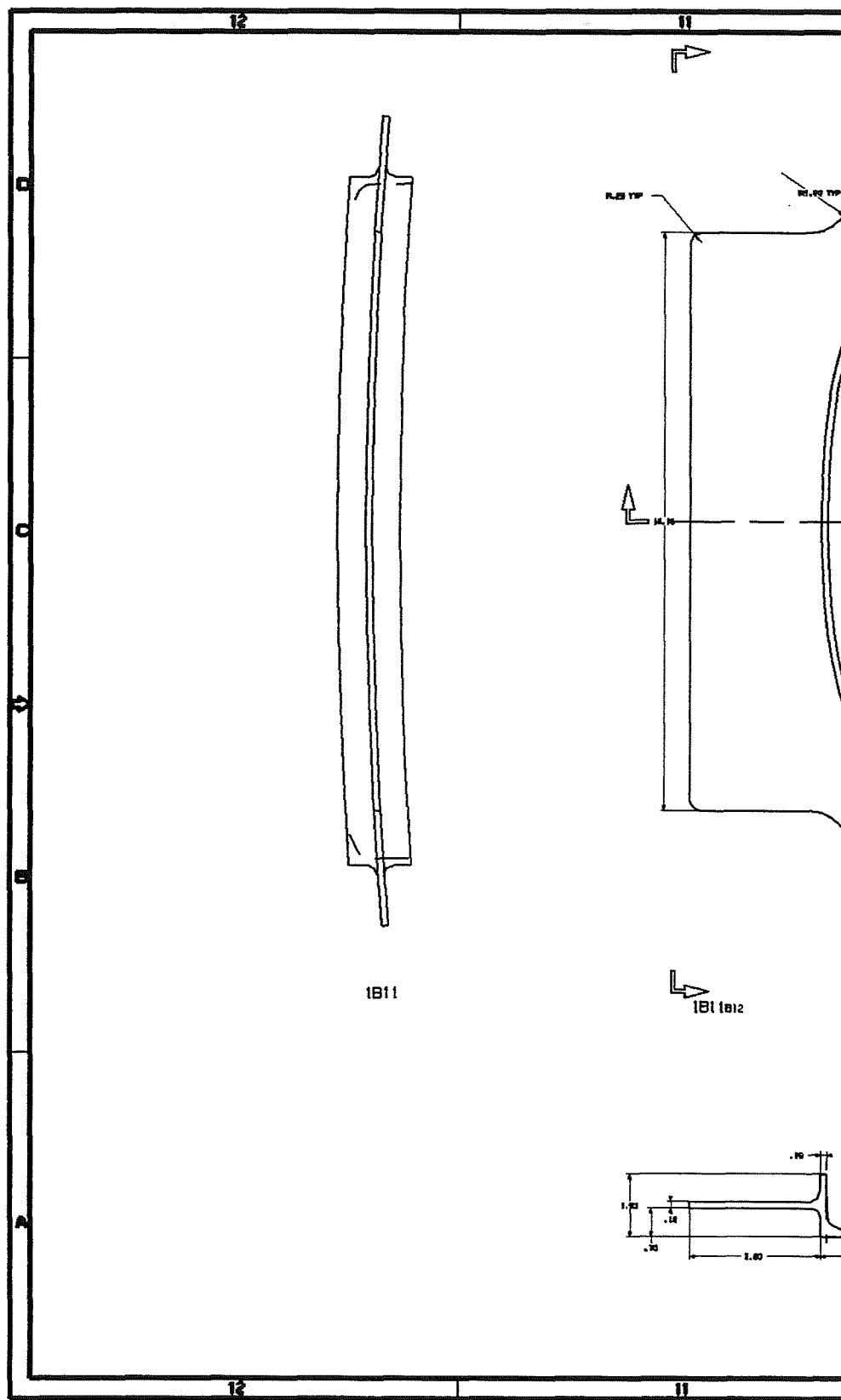
D-34

D-33

D-32

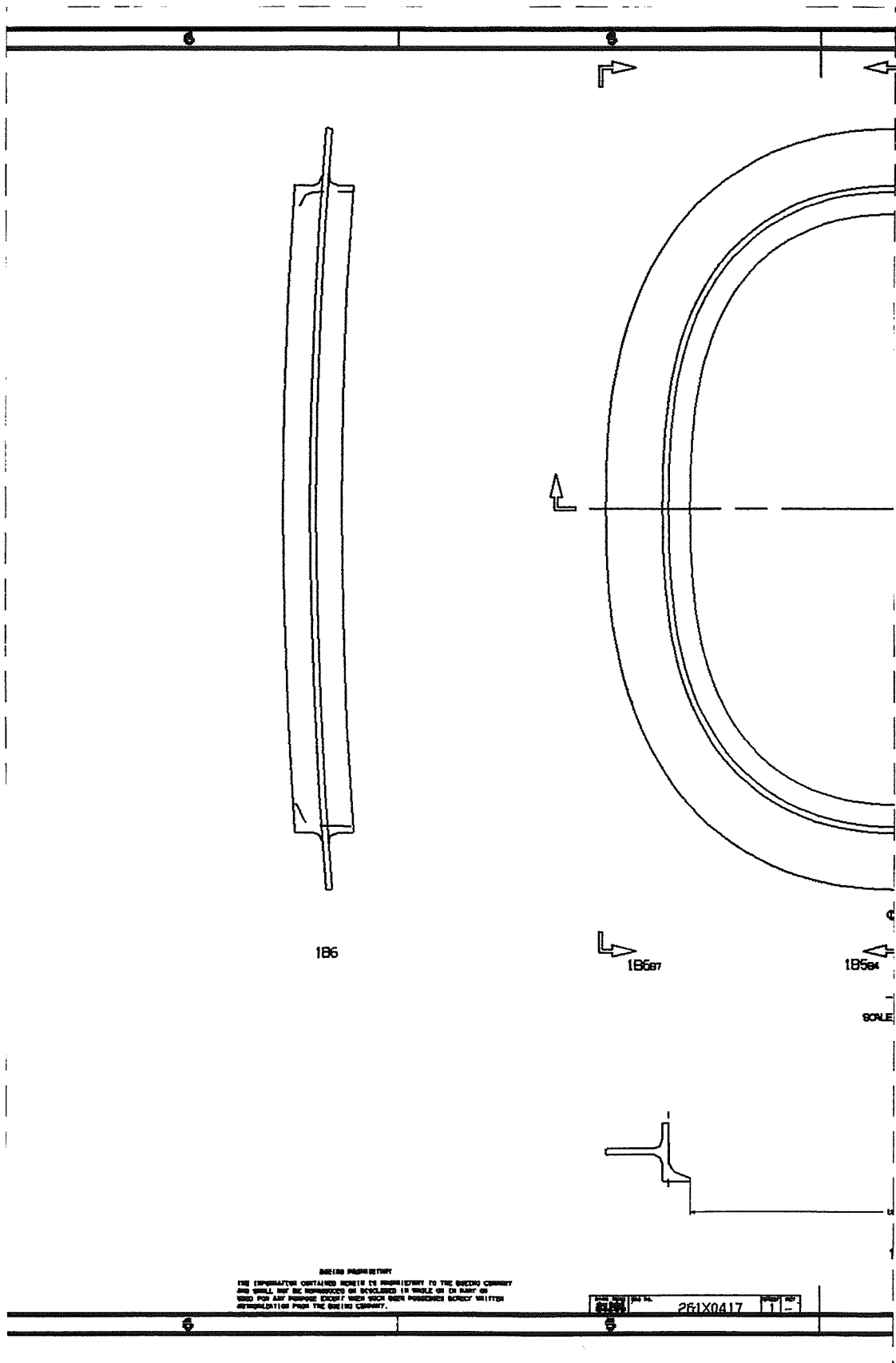
D-31

SP1X0013 1







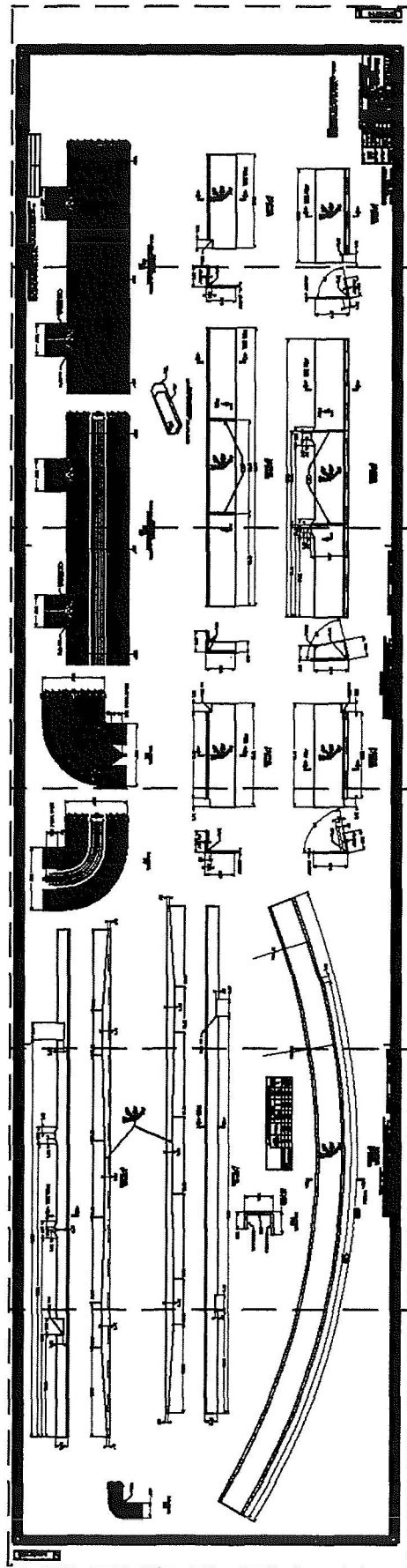


185

186

3D VIEW
ENGINEERING
SCALE: NOT TO SCALE

3D VIEW
ENGINEERING P
SCALE: NON



D-38

D-39

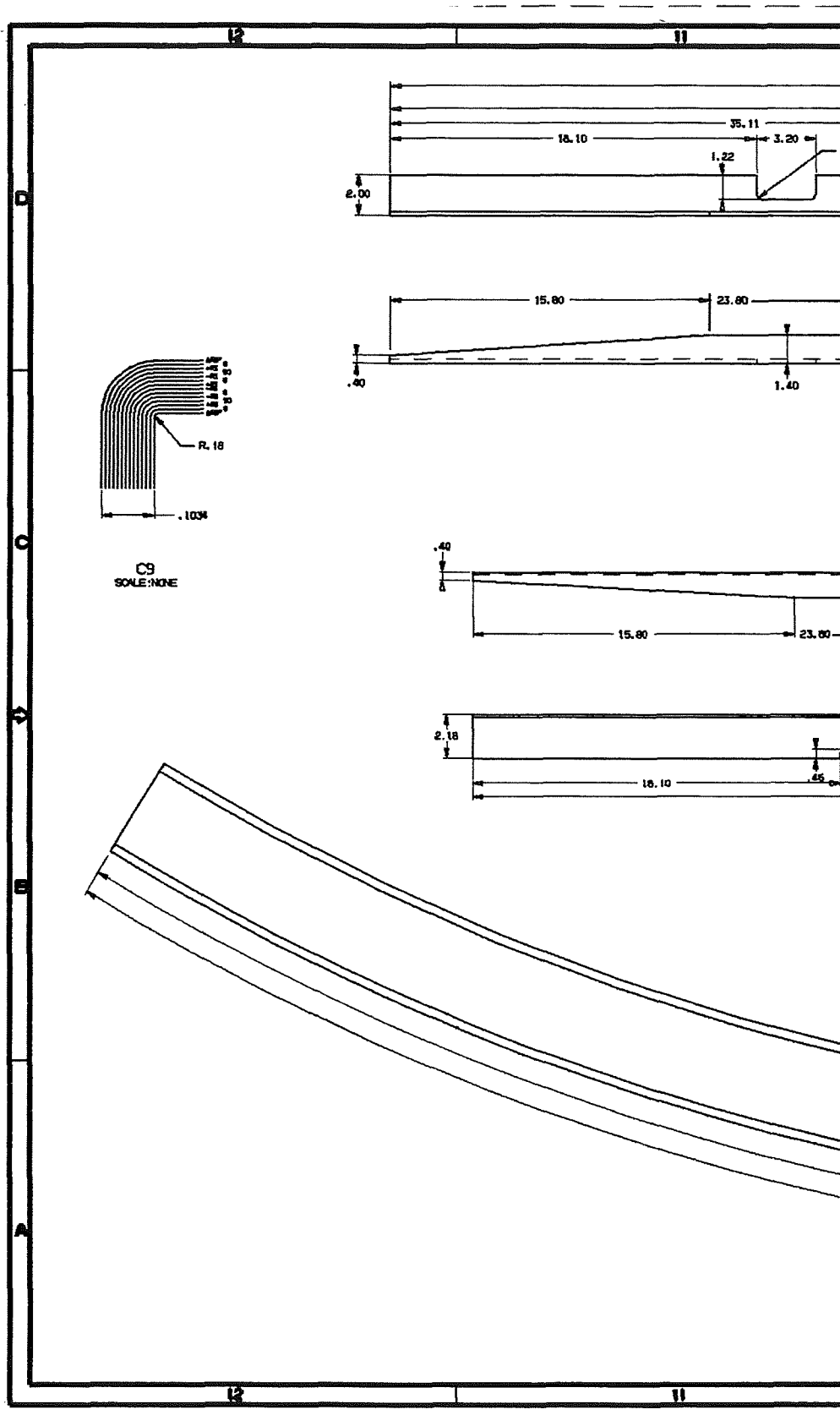
D-40

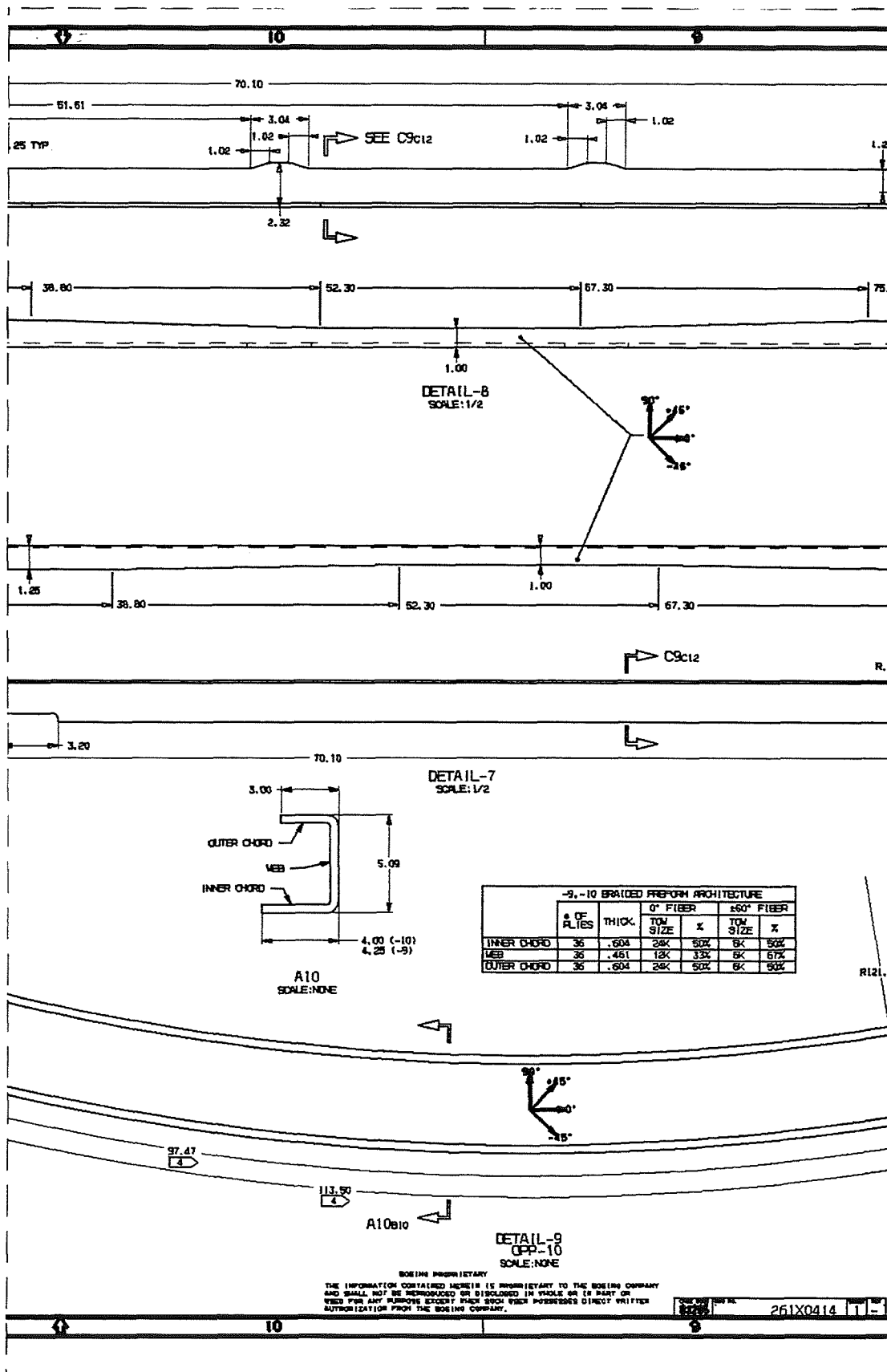
D-41

D-42

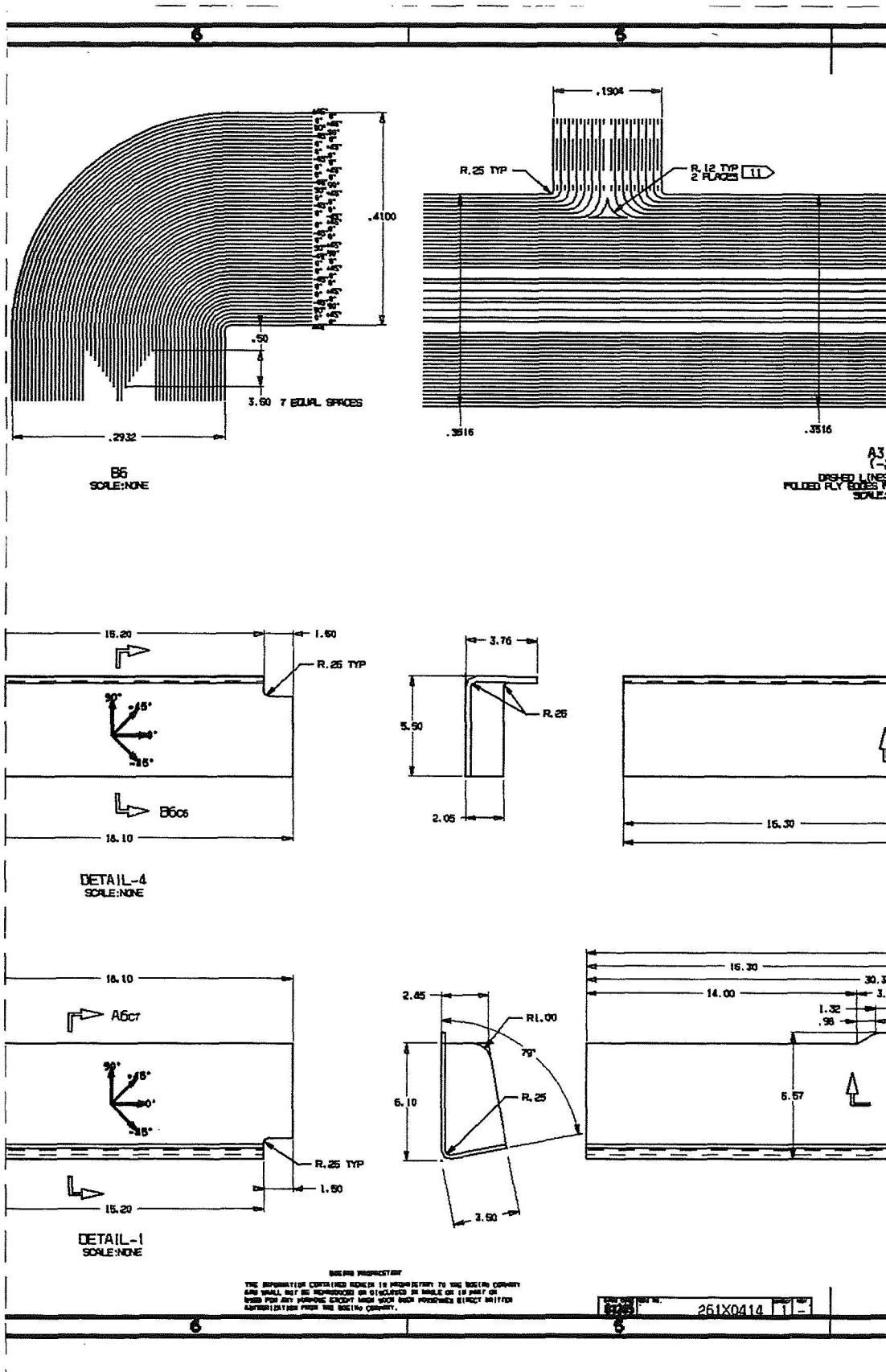
D-43

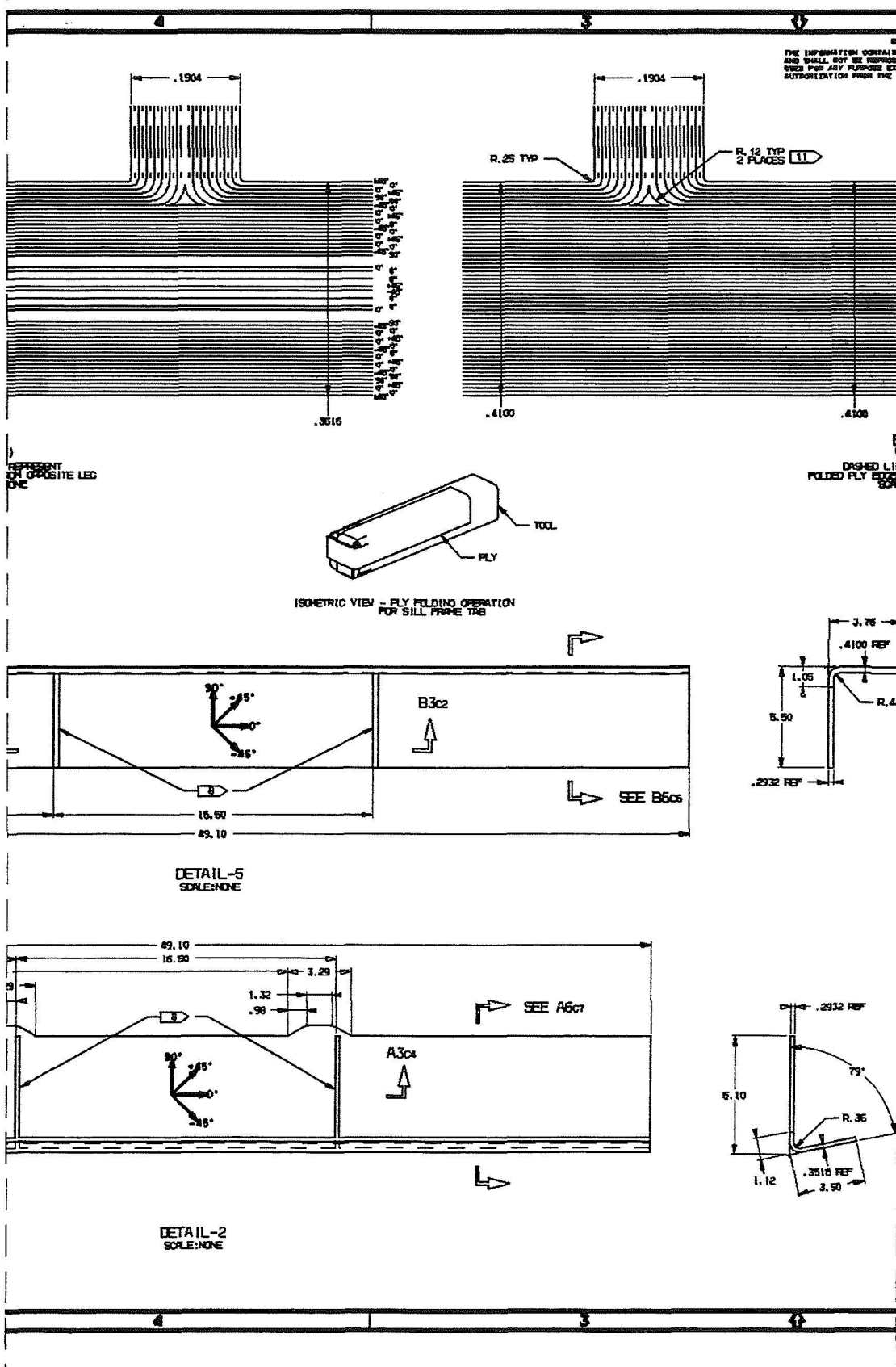
11 1140X1145

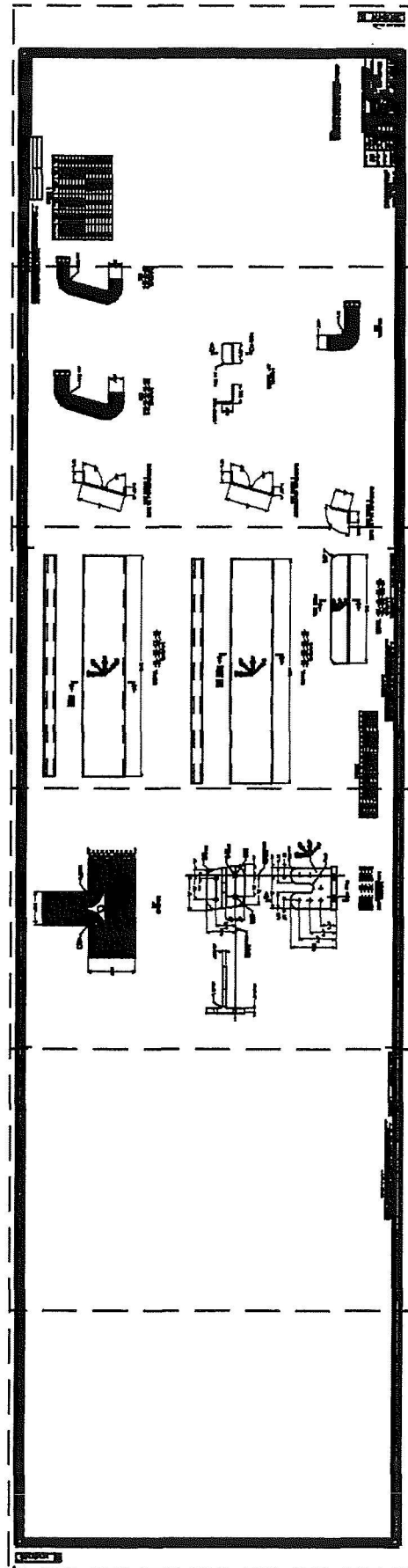












D-44

D-45

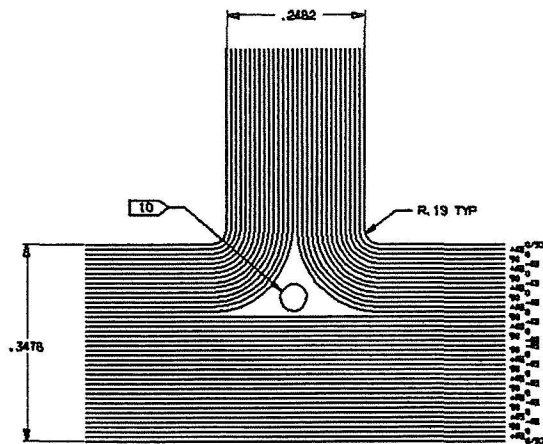
D-46

D-47

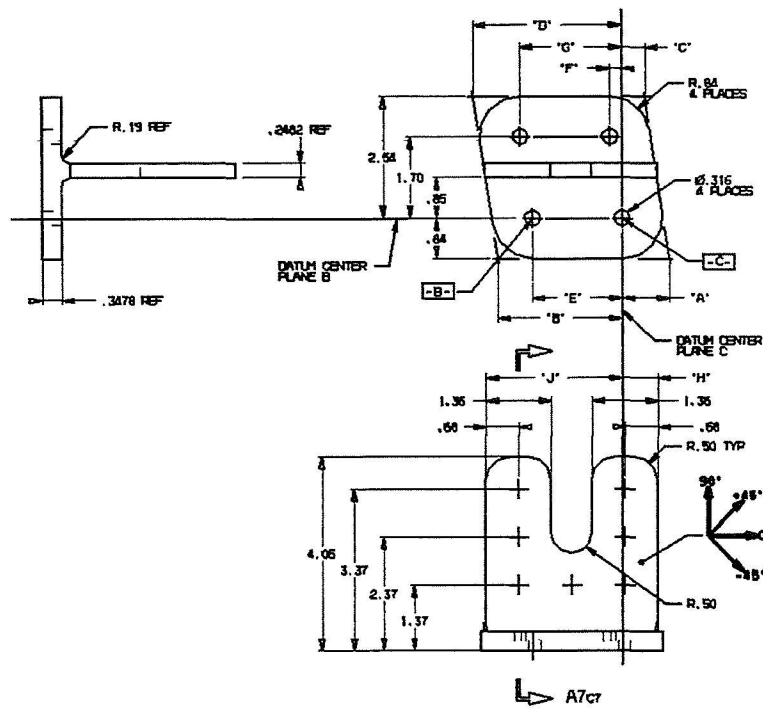
D-48

D-49

10	10	9
<p>BOEING PROPRIETARY</p> <p>THE INFORMATION CONTAINED HEREIN IS PROPRIETARY TO THE BOEING COMPANY AND SHALL NOT BE REPRODUCED OR DISCLOSED IN WHOLE OR IN PART OR USED FOR ANY PURPOSE EXCEPT UNDER SUCH OTHER FORMAL WRITTEN AUTHORIZATION FROM THE BOEING COMPANY.</p>		
10	10	9



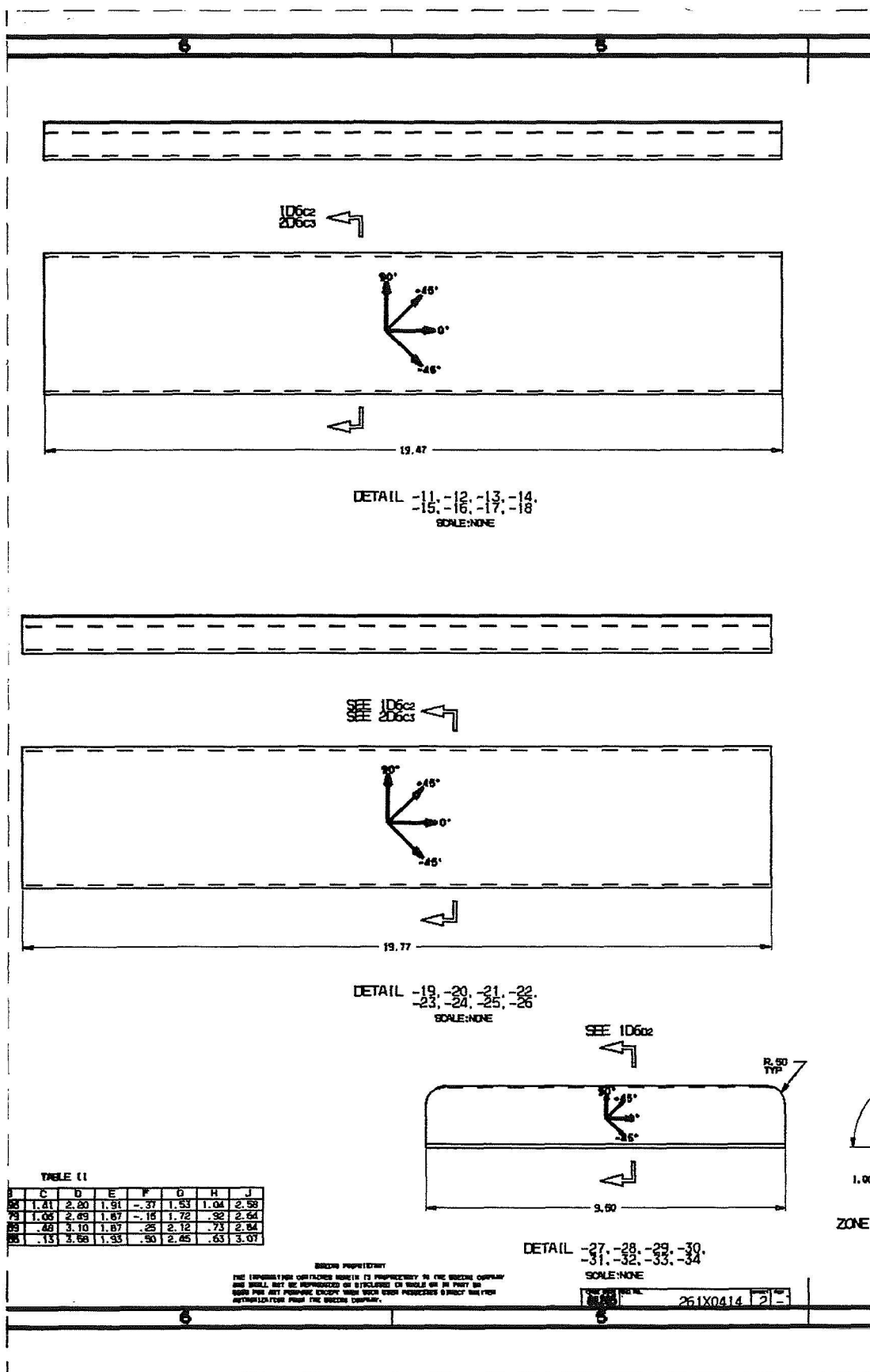
A7
SCALE: NONE



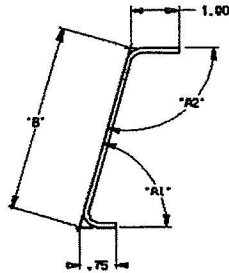
DETAIL -35, OFF-39
DETAIL -36, OFF-40
DETAIL -37, OFF-41
DETAIL -38, OFF-42

SEE TABLE II
ZONE A6 FOR DIMENSIONS
SCALE: NONE

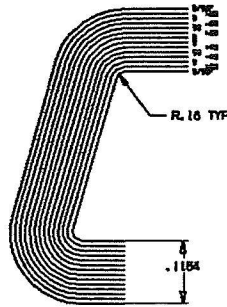
DIST #	A
-35, -39	.68 2
-36, -40	.77 2
-37, -41	.97 2
-38, -42	1.12 2



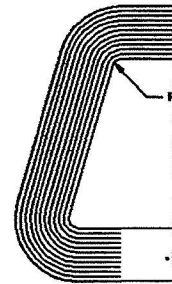
THE INFORMATION CONTAINED
HEREIN IS UNCLASSIFIED
DATE 10-15-2010 BY 60322
AUTHORITY 101-101



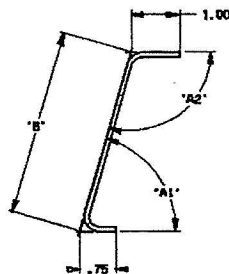
SEE TABLE I
ZONE D2 FOR DIMENSIONS



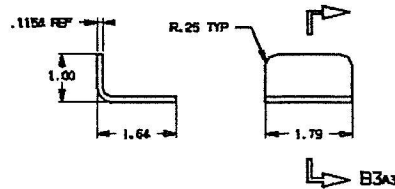
206
(-12, -13, -14, -15, -17)
(-20, -21, -22, -23, -25)
SCALE: NONE



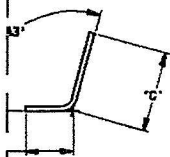
106
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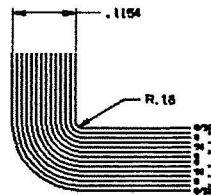
SEE TABLE I
ZONE D2 FOR DIMENSIONS



DETAIL -47
SCALE: 1/1



SEE TABLE I
ZONE D2 FOR DIMENSIONS



B3
SCALE: NONE

NO PROPRIETARY		REVISED			
HEREIN IS PROPRIETARY TO THE BOEING COMPANY AND IS NOT TO BE DISCLOSED IN WHOLE OR IN PART OR BY ANY MEANS OR BY ANY PROCESS DIRECT OR INDIRECT TO THE COMPANY.		DESCRIPTION	DATE		
TABLE 1					
DRG NO.	ANGLE A1°	ANGLE A2°	ANGLE A3°	DIM B	DIM C
-11-19	73.36°	105.04°	---	4.16	---
-12-20	77.51°	102.39°	---	4.12	---
-13-21	81.26°	98.74°	---	4.07	---
-14-22	84.91°	95.09°	---	4.04	---
-15-23	91.66°	88.14°	---	4.02	---
-16-24	98.07°	81.93°	---	4.07	---
-17-25	102.10°	77.90°	---	4.12	---
-18-26	106.14°	73.66°	---	4.16	---
-27	---	---	105.56°	---	1.57
-28	---	---	101.92°	---	1.54
-29	---	---	96.29°	---	1.51
-30	---	---	94.65°	---	1.53
-31	---	---	87.74°	---	1.55
-32	---	---	81.55°	---	1.53
-33	---	---	77.54°	---	1.52
-34	---	---	73.52°	---	1.51

NOTE:
SOLIDS IN DATASET ARE FOR PICTORIAL
PURPOSE ONLY. THEY DO NOT ACCURATELY REFLECT
DIMENSIONS AS SHOWN ON DRG SHEET.

SEE PL FOR LIST OF MATERIAL, USAGE AND NOTES

USED IN	GROUP	TURN LE	DATE	BOEING
TEST ONLY	GROUP 1	C. HENCH	11/15/75	COMPOSITE SANDWICH
TEST ONLY	GROUP 2	C. HENCH	11/23/75	SIDE PANEL - DOOR DETAILS
TEST ONLY	GROUP 3	C. HENCH	11/23/75	(TEST ONLY)
TEST ONLY	GROUP 4	D. SCHULTZ	11/23/75	---
TEST ONLY	GROUP 5	P. SMITH	11/26/75	---
TEST ONLY	GROUP 6	P. SMITH	11/26/75	---
TEST ONLY	GROUP 7	S. METZGER	11/22/75	---
TEST ONLY	GROUP 8	J. HOFFMANN	11/22/75	---

UNLESS OTHERWISE SPECIFIED
DIMENSIONS ARE IN INCHES

TOLERANCES:

ANGLES ± .03

B-7642

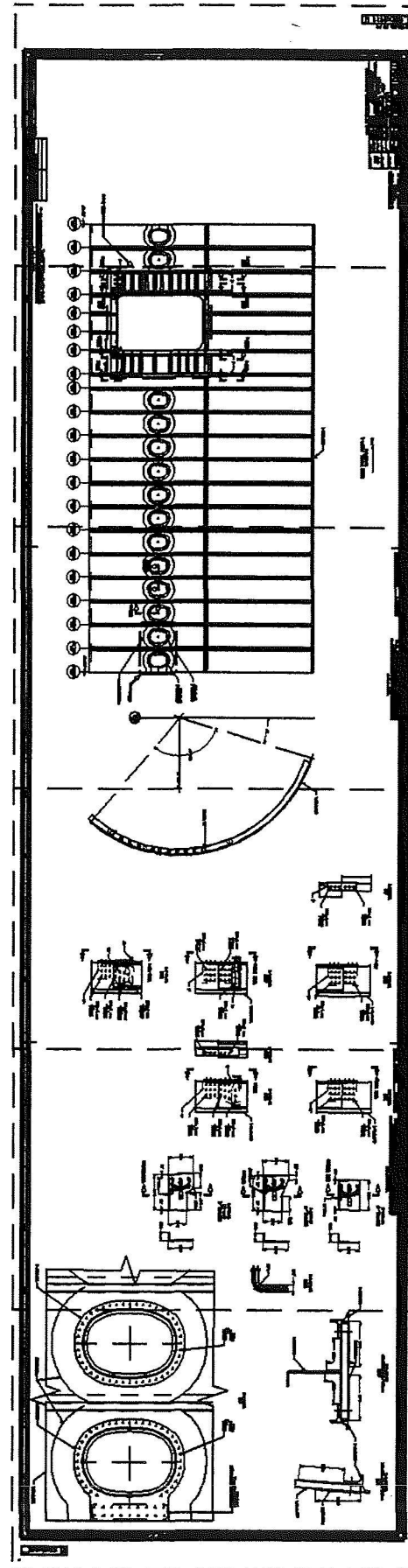
261X0414

J 81205

261X0414

DRAWING RECORDS

2



D-51

D-52

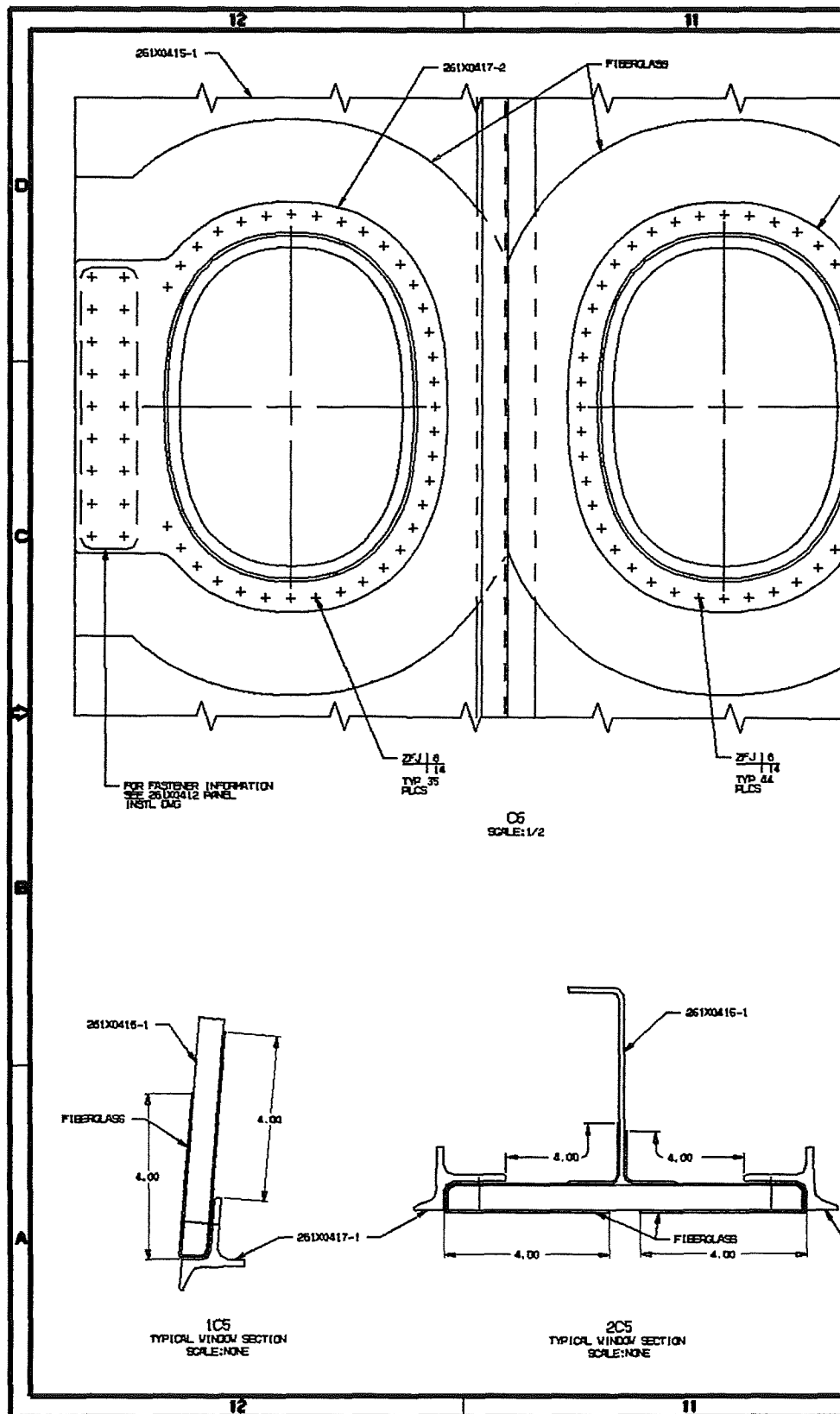
D-53

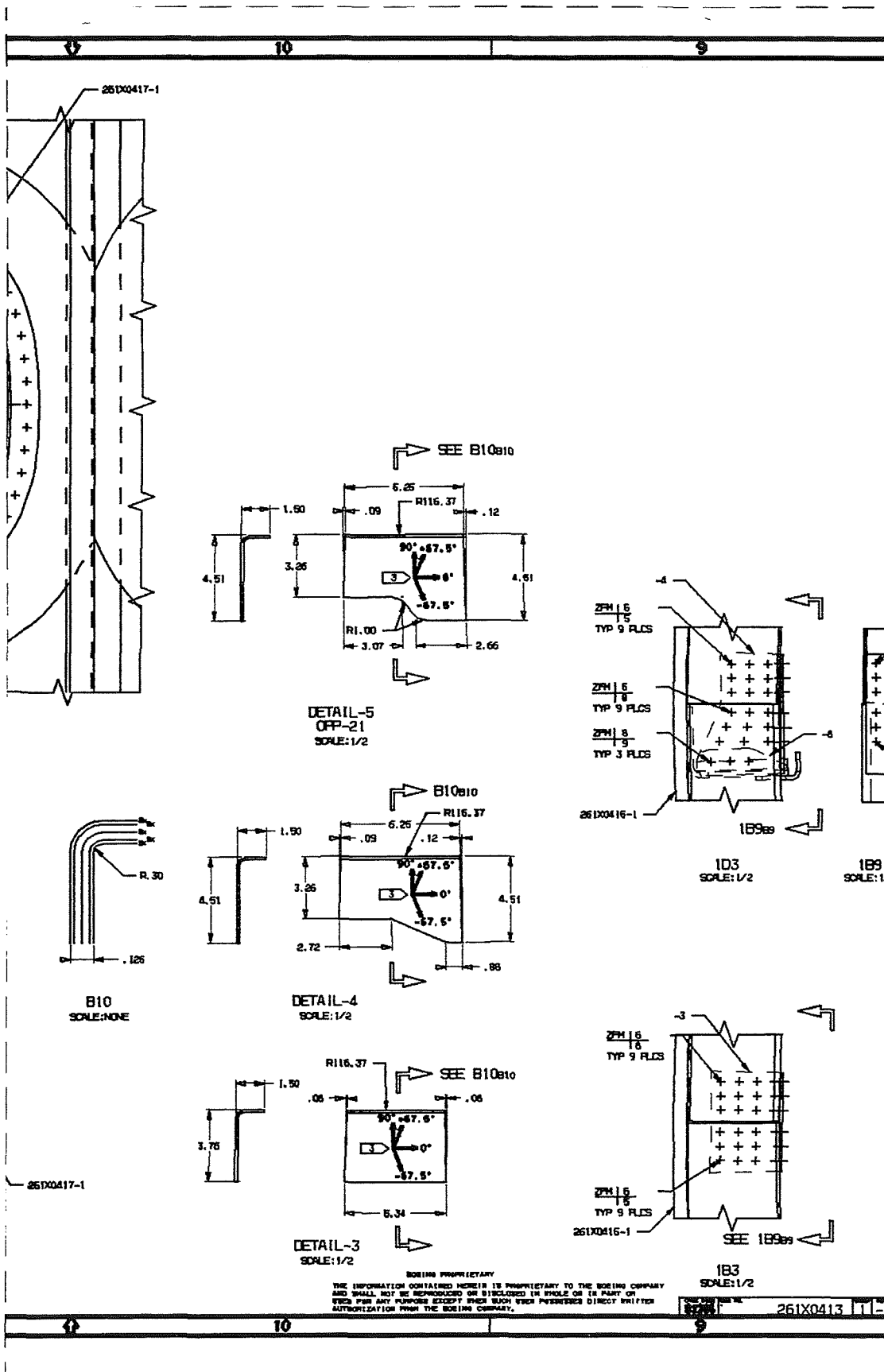
D-54

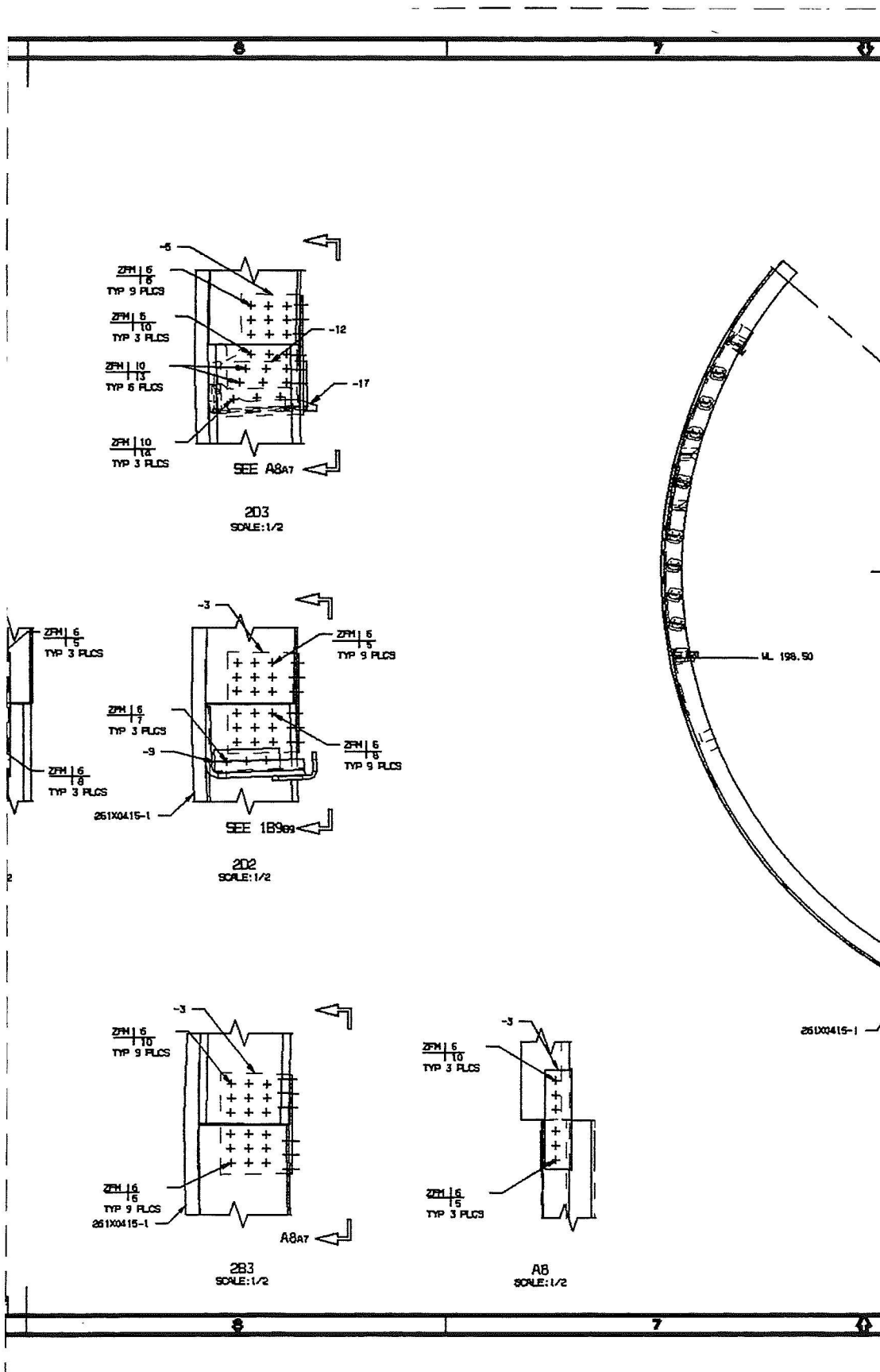
D-55

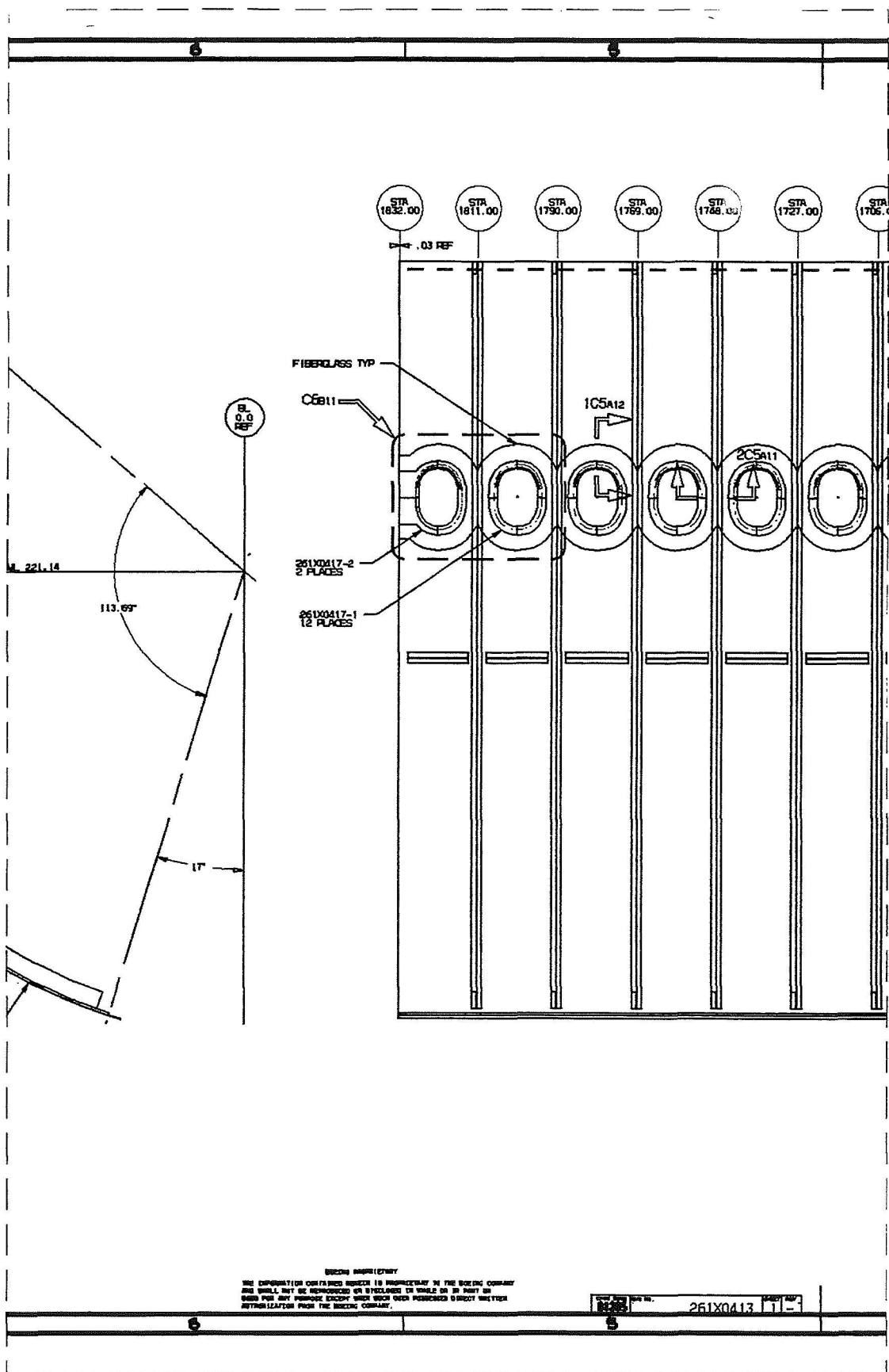
D-56

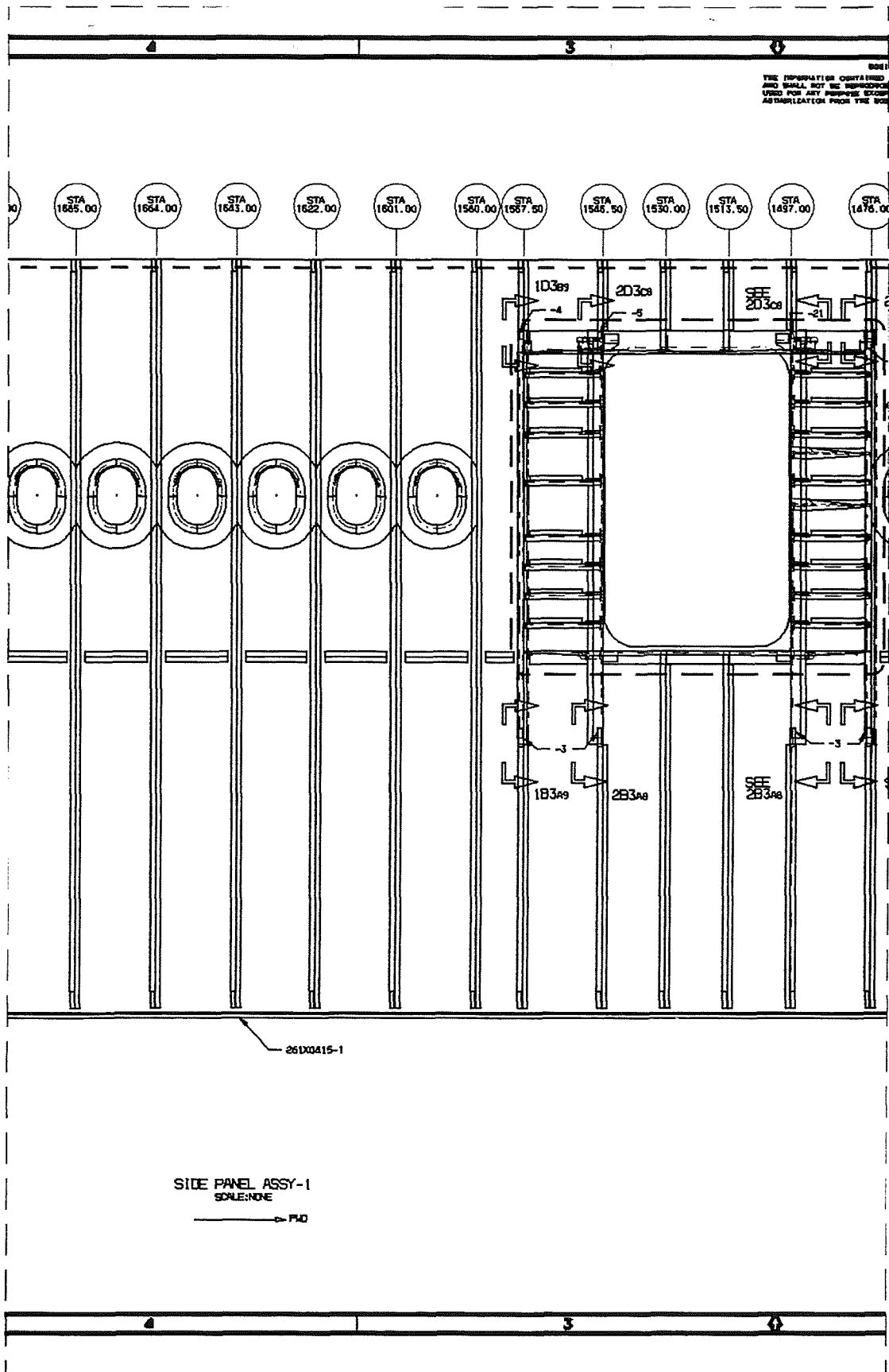
SP1X0413 1



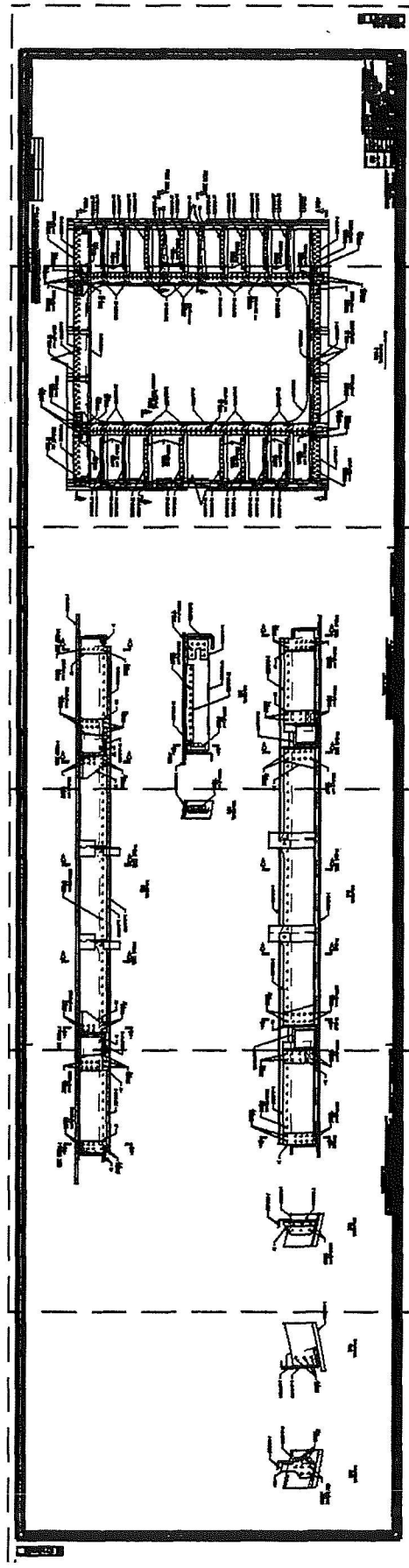








D-57



D-58

D-59

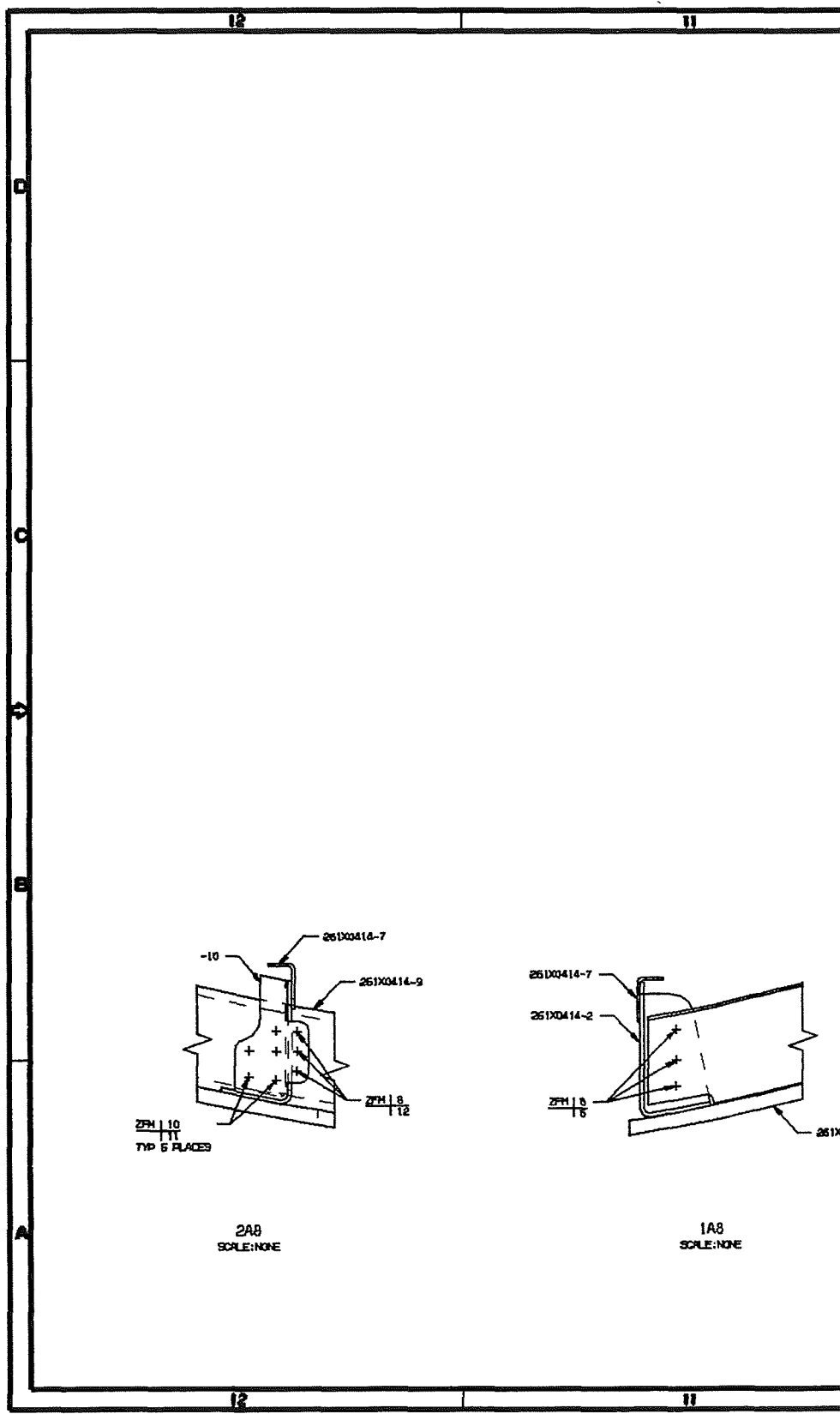
D-60

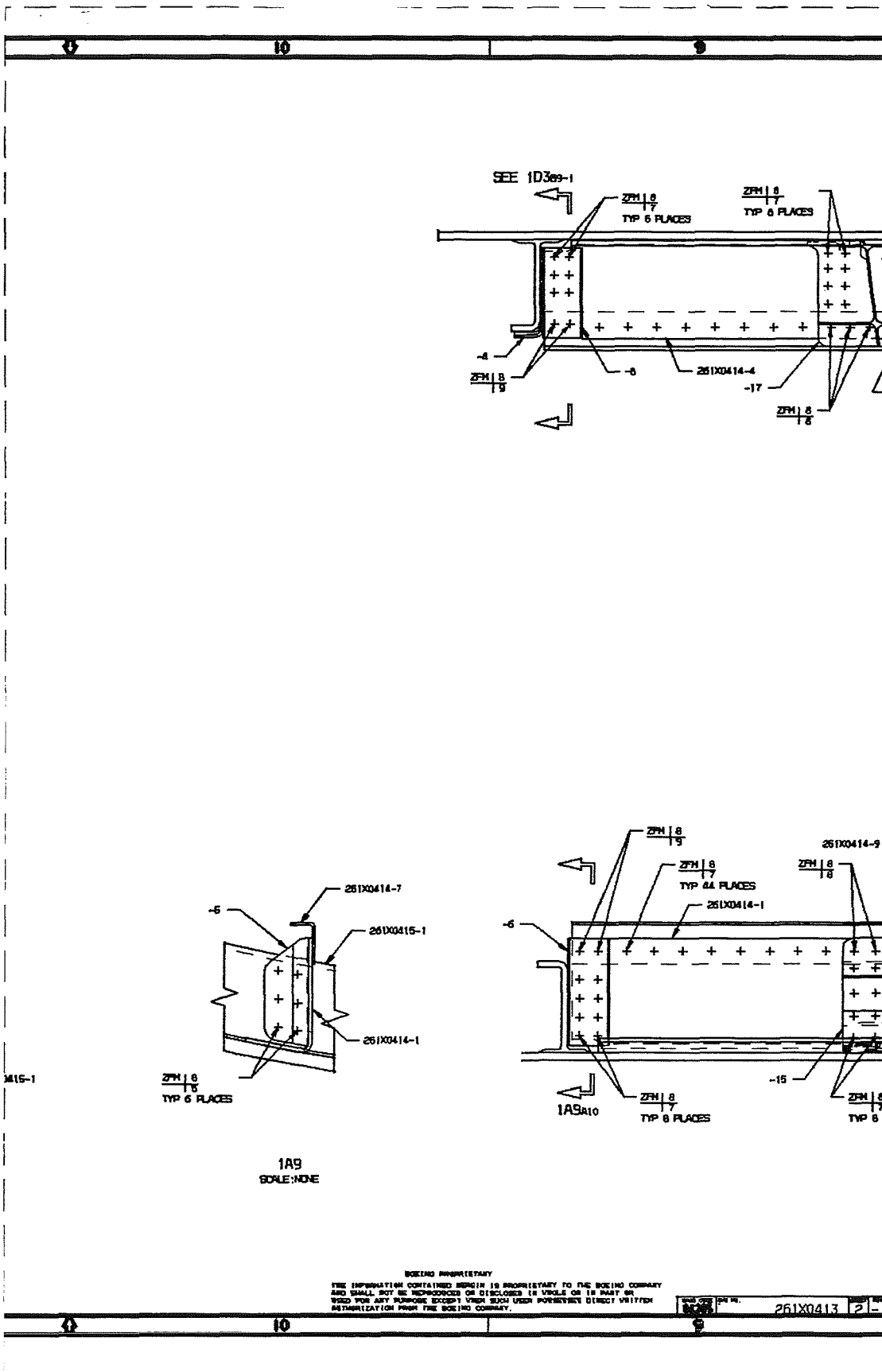
D-61

D-62

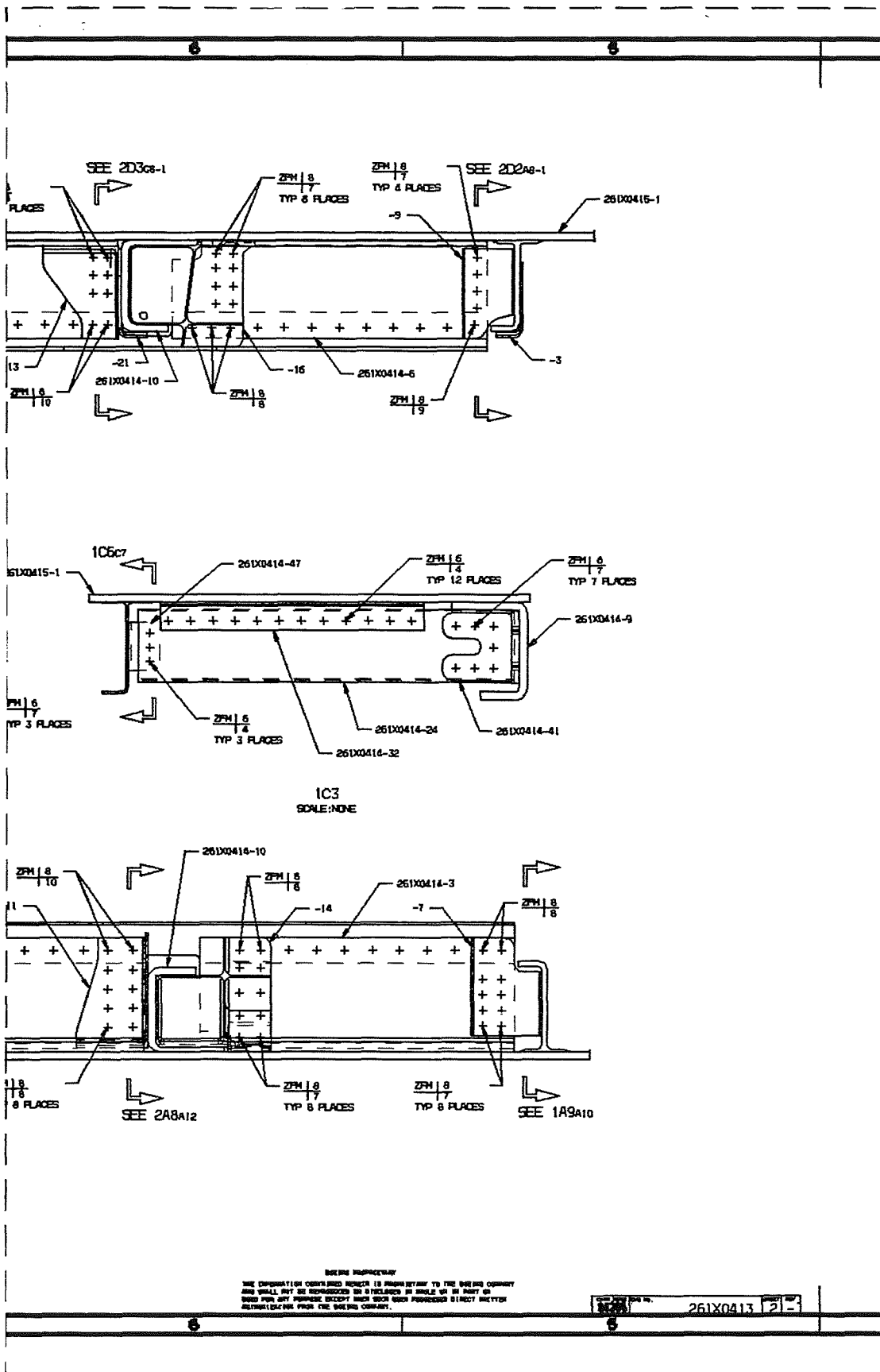
D-63

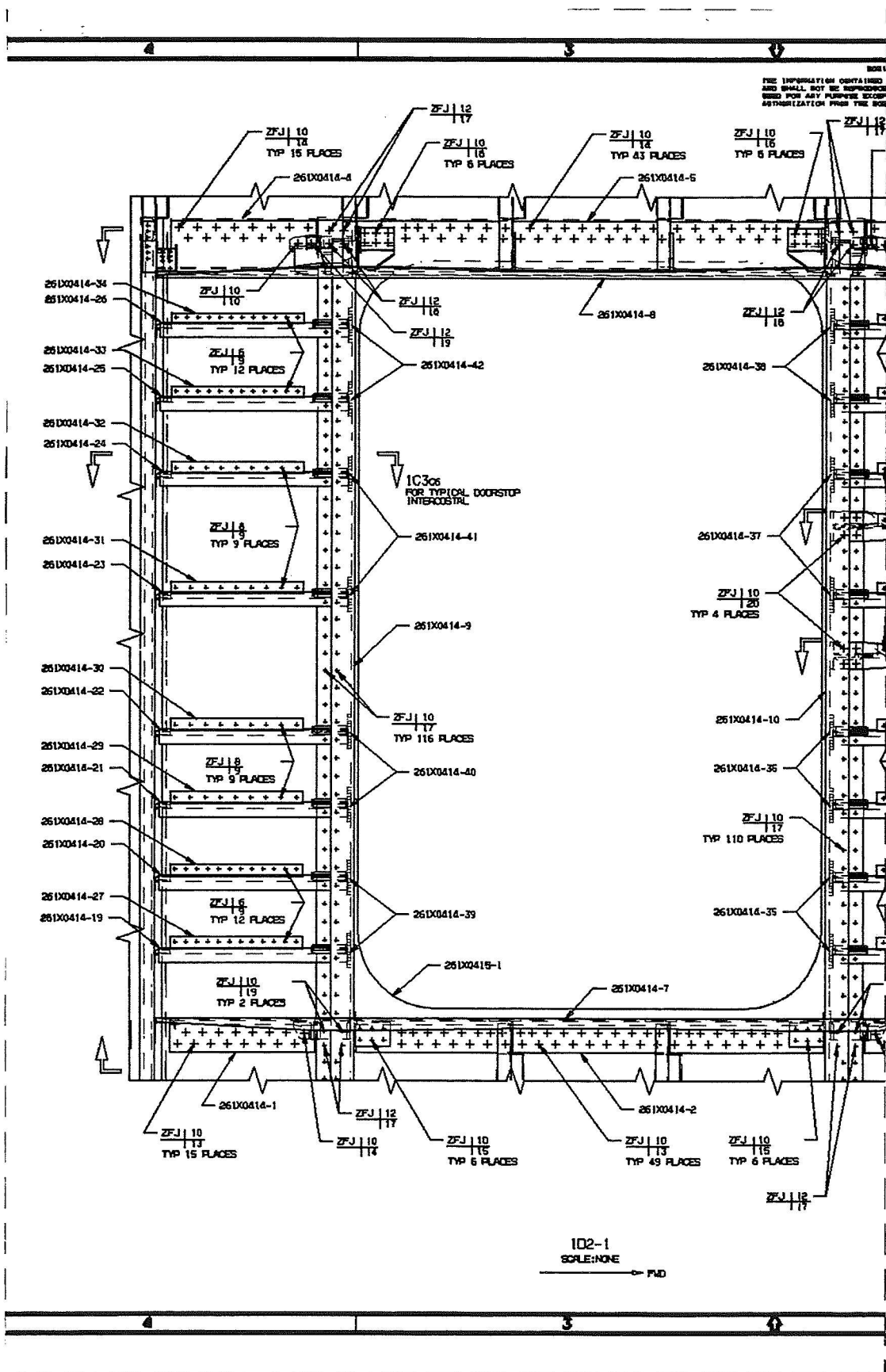
261X0413 S

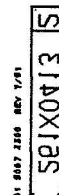


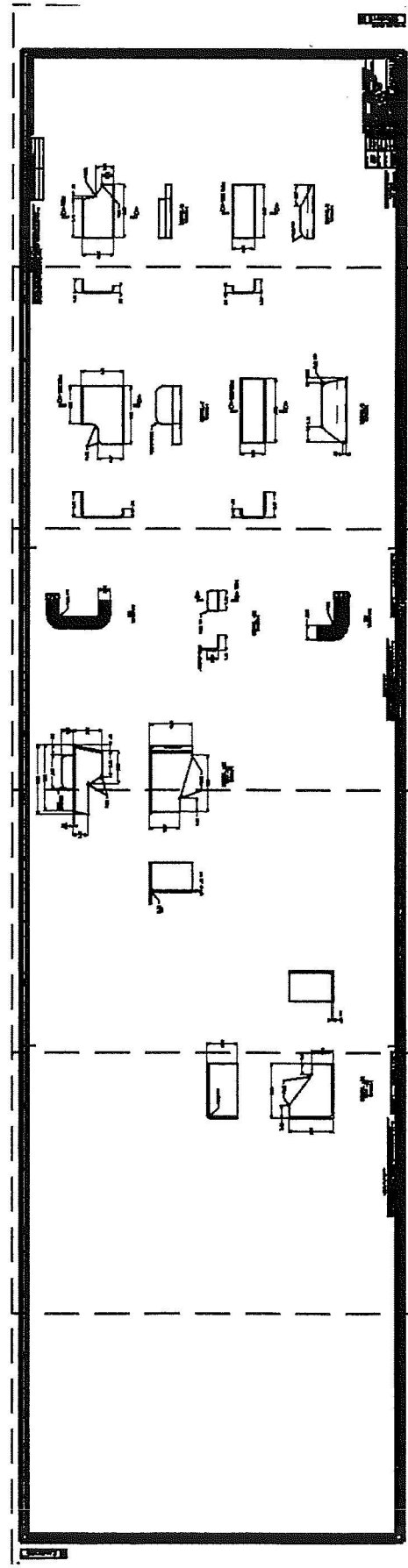












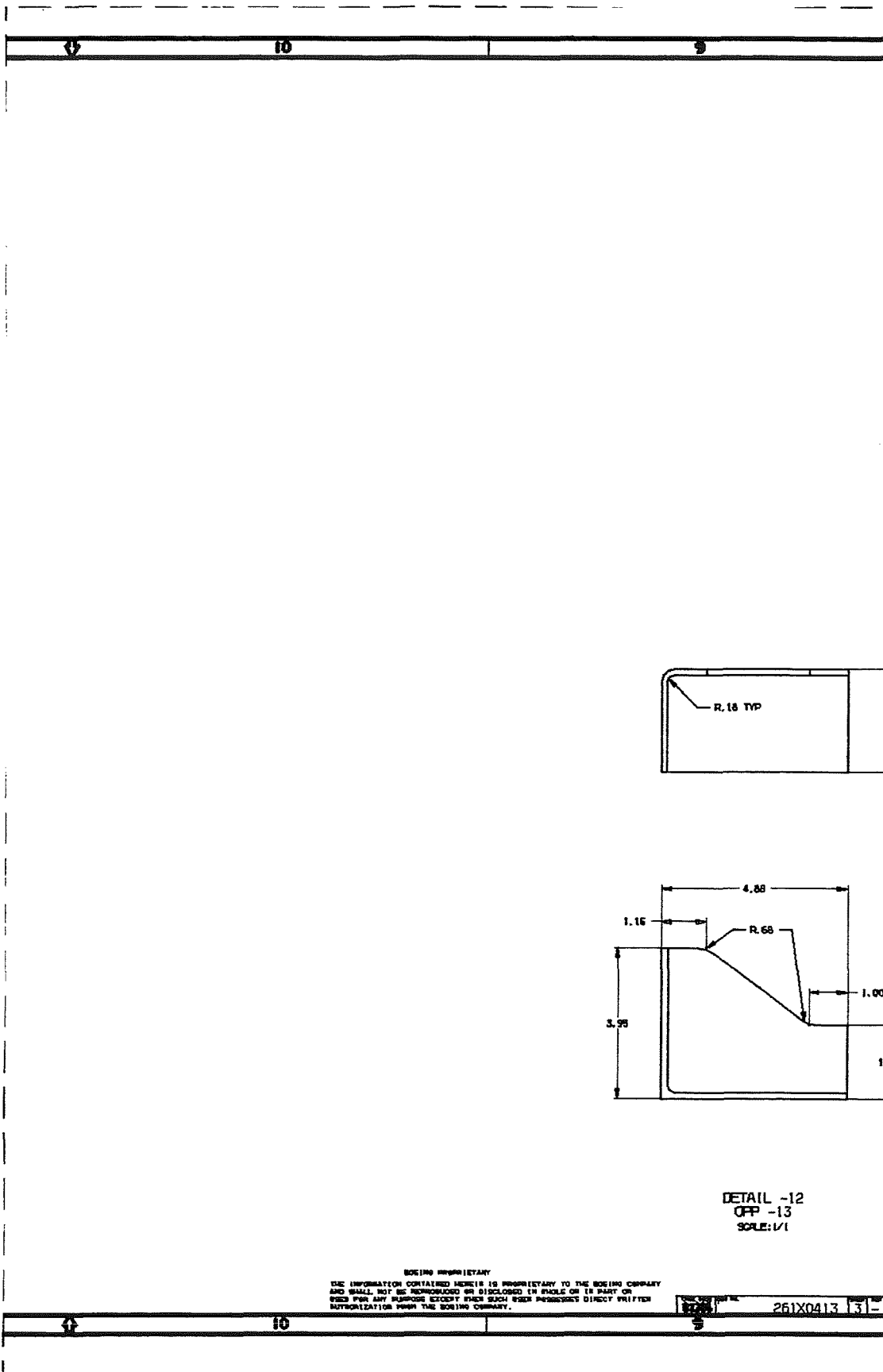
D-65

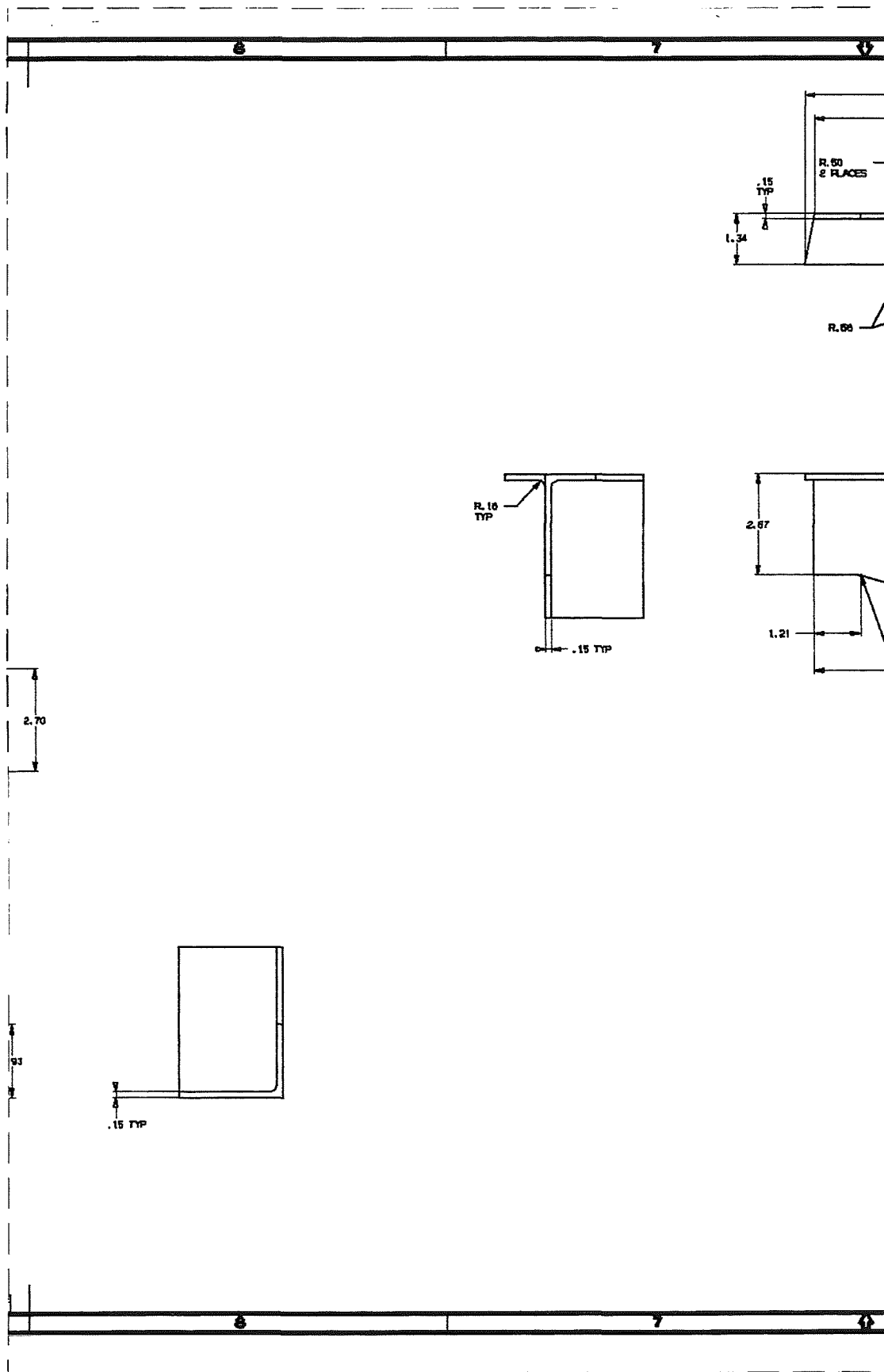
D-66

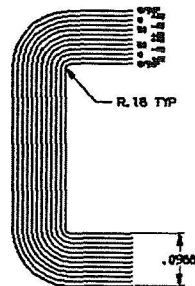
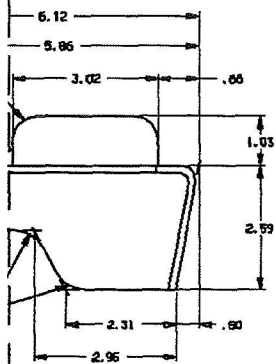
D-67

D-68

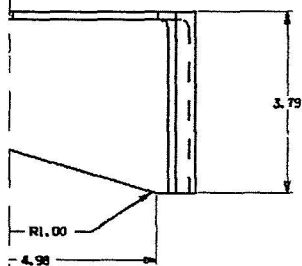
D-69



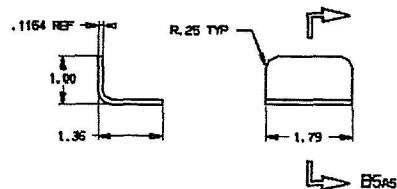




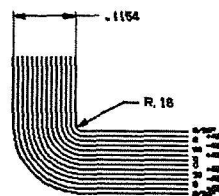
D2
SCALE:NONE



DETAIL -10
OFF -11
SCALE:1/1



DETAIL -20
SCALE:1/1

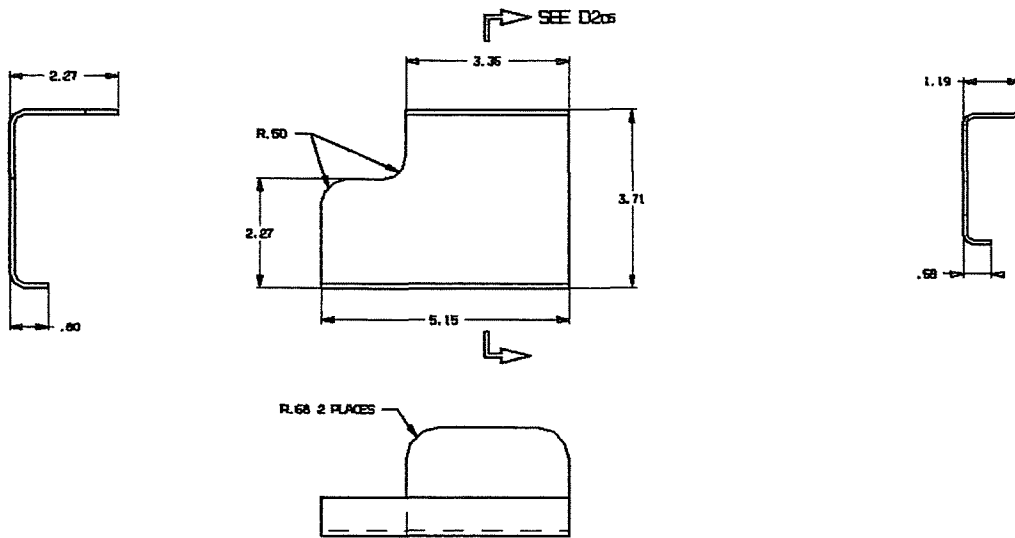


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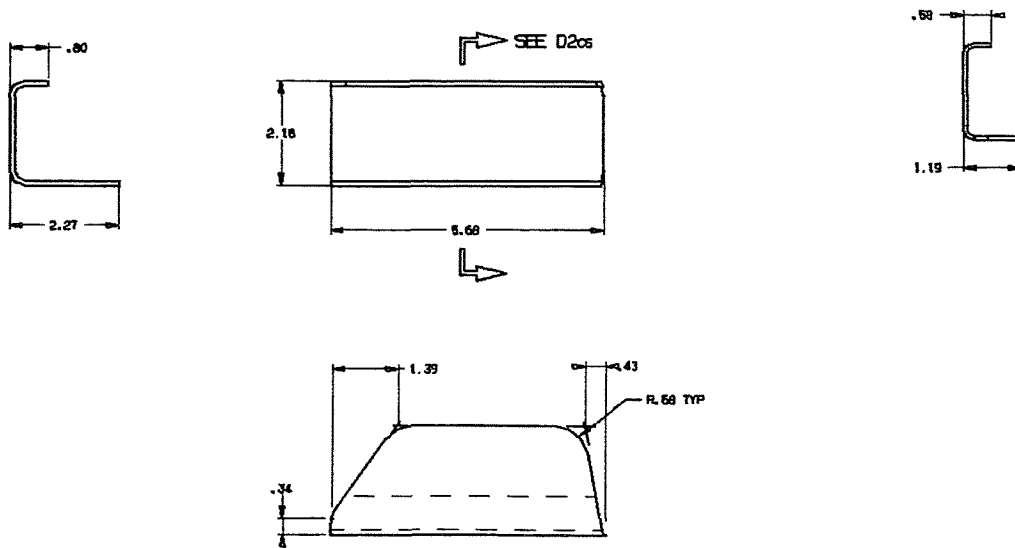
BUSSING PROPRIETARY
THE INFORMATION CONTAINED HEREIN IS PROPRIETARY TO THE BUSSING COMPANY
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USED FOR ANY PURPOSES EXCEPT ASH FOR USES INDICATED DIRECT WRITTEN
AUTHORIZATION FROM THE BUSSING COMPANY.

261X0413 3

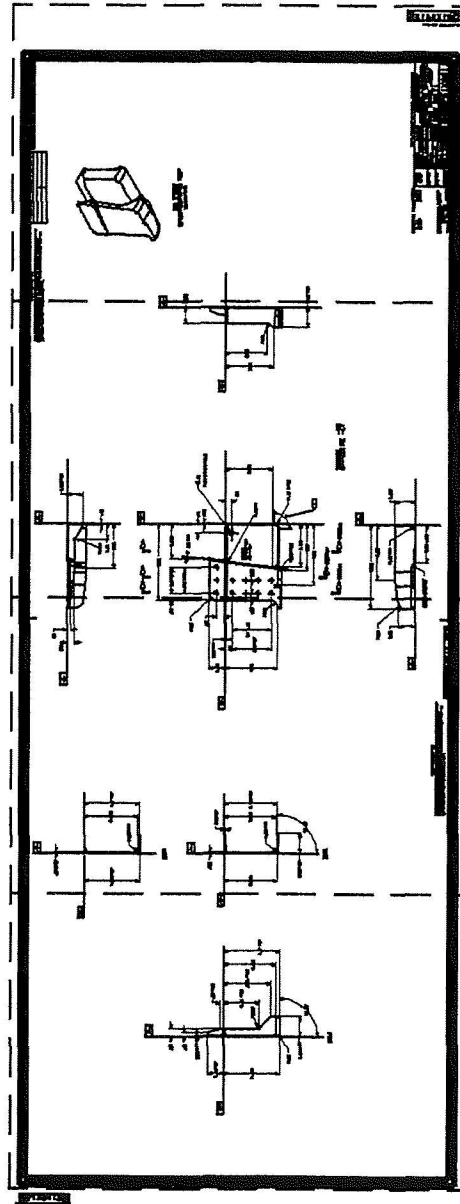
THIS INFORMATION CONTAINED
HEREIN IS UNCLASSIFIED
DATE 08-11-2010 BY 60322
AUTHORITY 5010-108



DETAIL-7
SCALE: 1/1



DETAIL-6
SCALE: 1/1

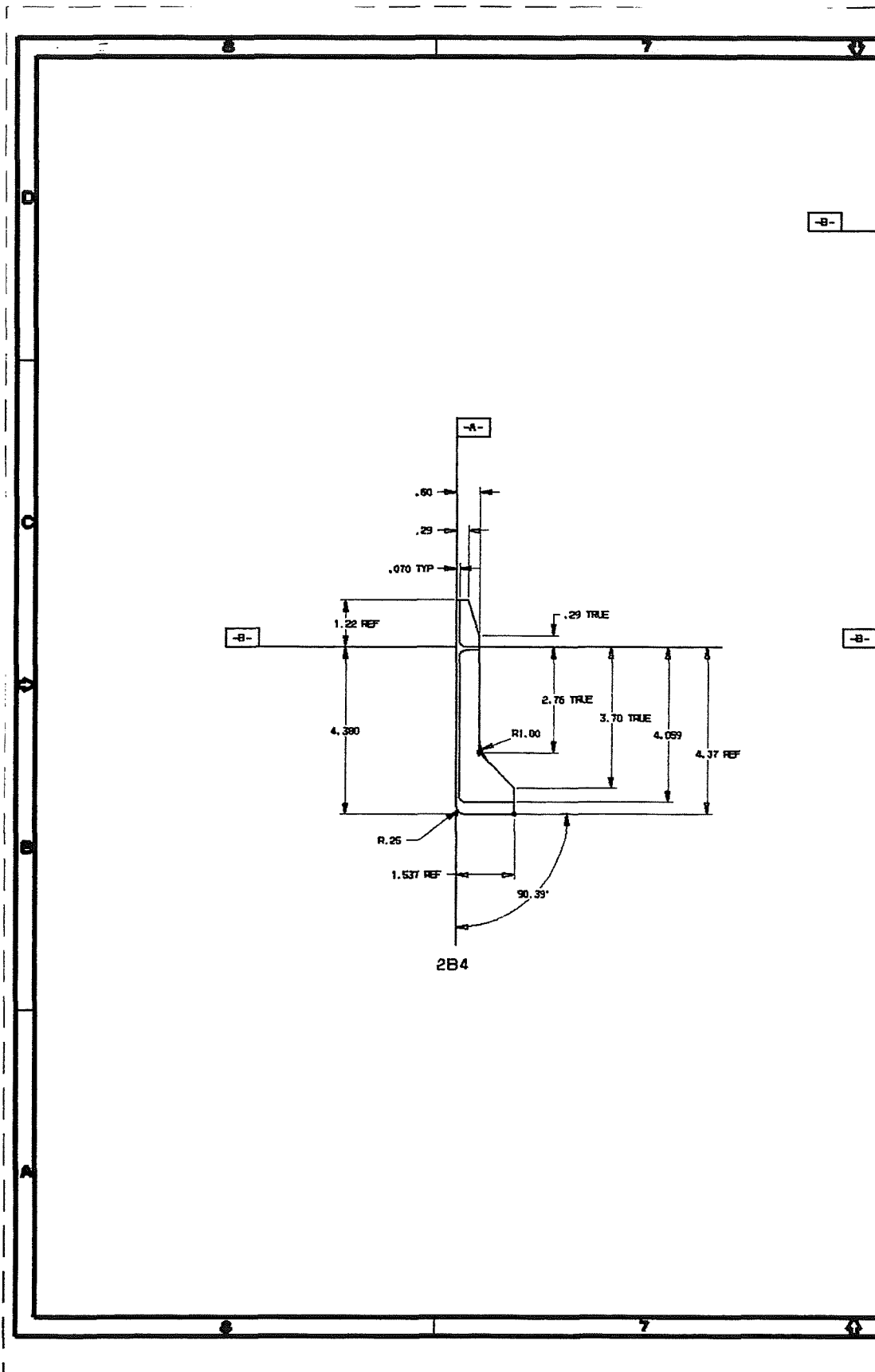


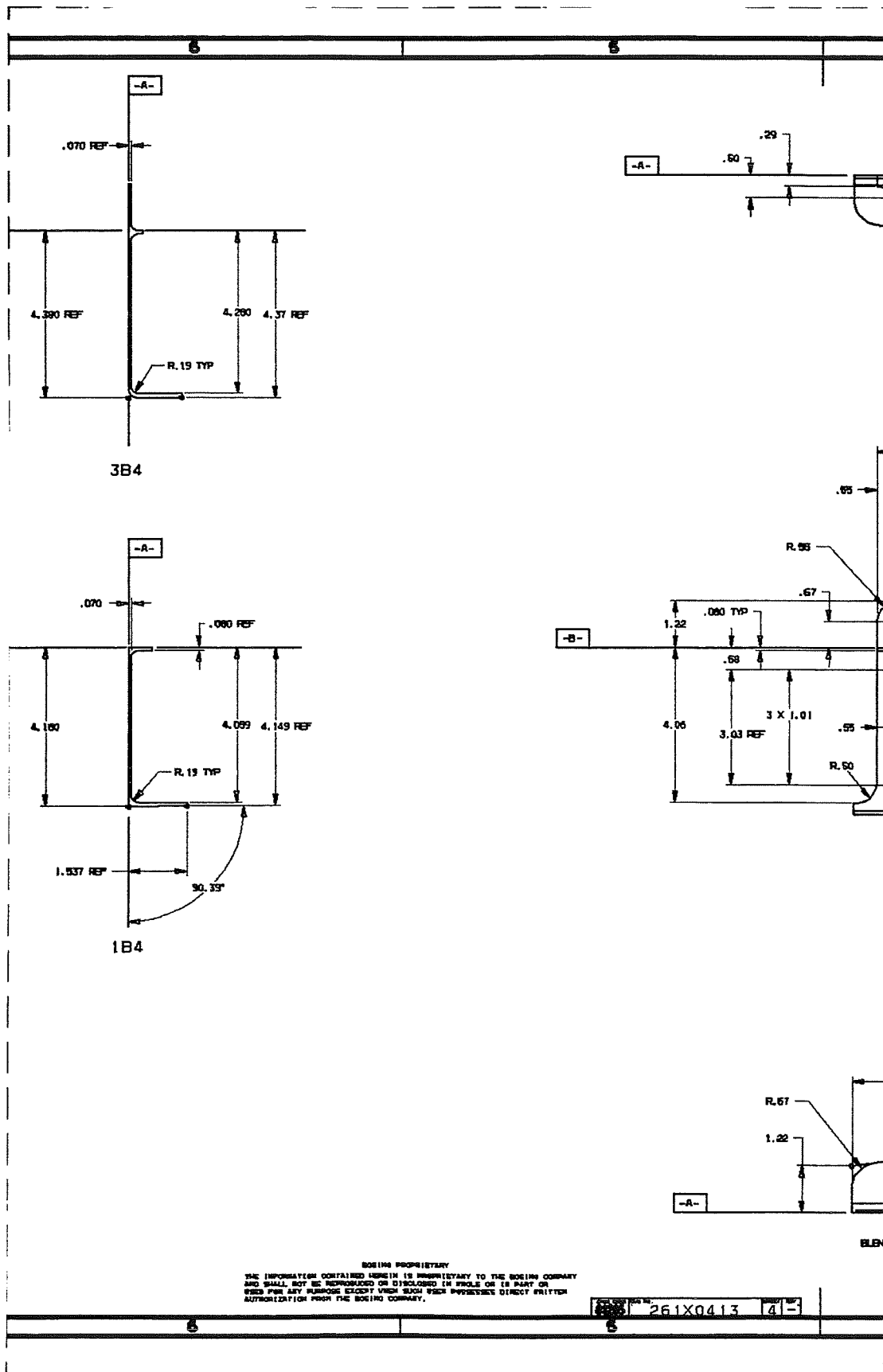
D-71

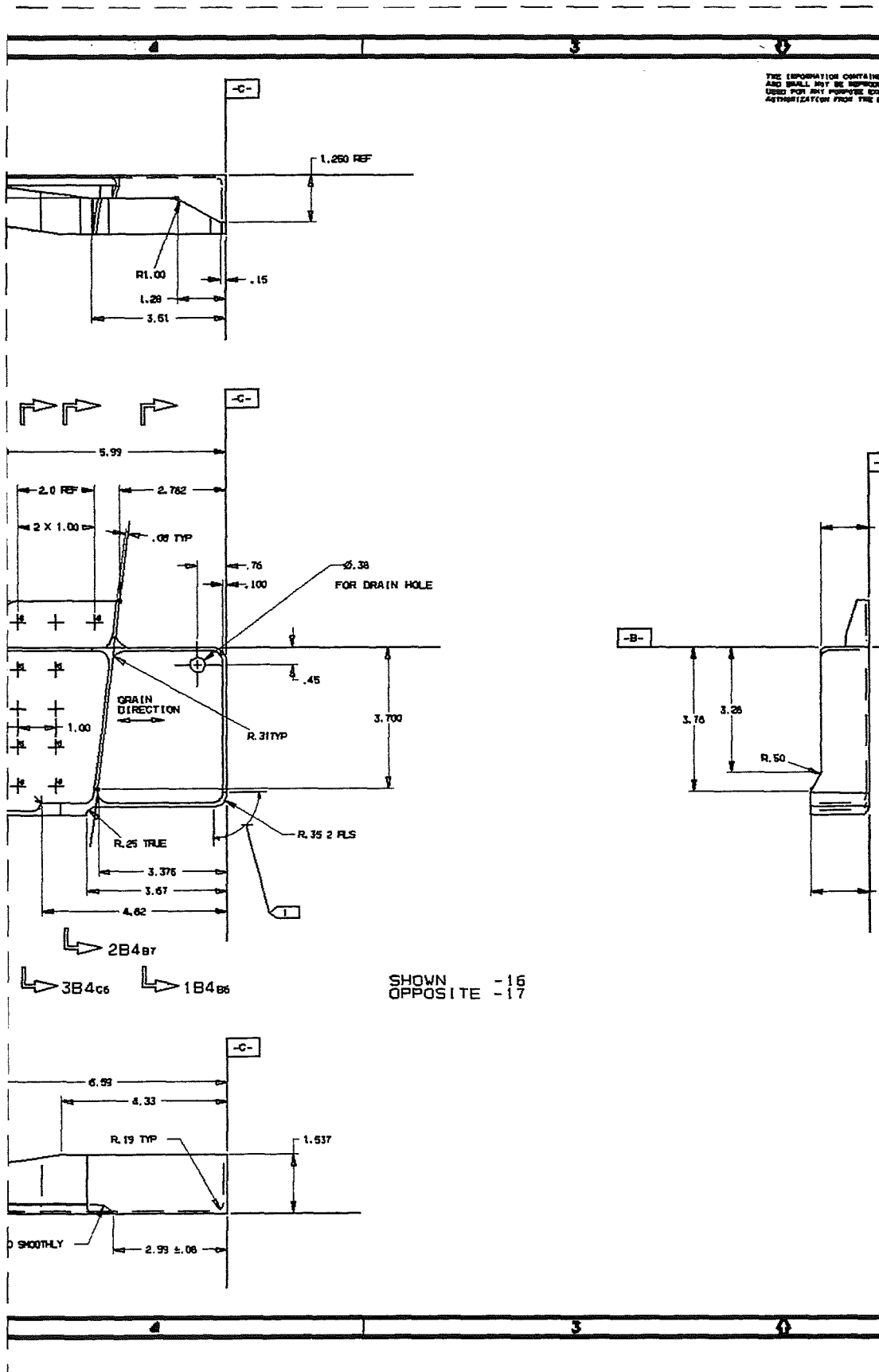
D-72

D-73

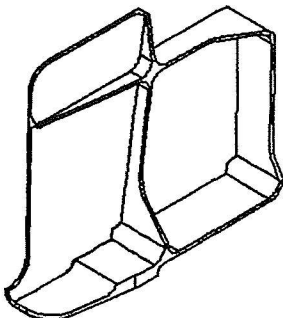
D-74







BOEING PROPRIETARY THIS DRAWING IS PROPRIETARY TO THE BOEING COMPANY AND IS DISCLOSED IN WHOLE OR IN PART OR BY ANY MANNER SUCH AS BY DIRECT WRITTEN OR ORAL COMPANY.	<table border="1" style="width: 100%; border-collapse: collapse;"> <tr> <td style="width: 50%;">DESIGN</td> <td style="width: 50%;">REVISIONS</td> </tr> <tr> <td> </td> <td> </td> </tr> <tr> <td> </td> <td> </td> </tr> <tr> <td> </td> <td> </td> </tr> </table>	DESIGN	REVISIONS							<table border="1" style="width: 100%; border-collapse: collapse;"> <tr> <td style="width: 50%;">DATE</td> <td style="width: 50%;">APPROVED</td> </tr> <tr> <td> </td> <td> </td> </tr> <tr> <td> </td> <td> </td> </tr> <tr> <td> </td> <td> </td> </tr> </table>	DATE	APPROVED						
DESIGN	REVISIONS																	
DATE	APPROVED																	



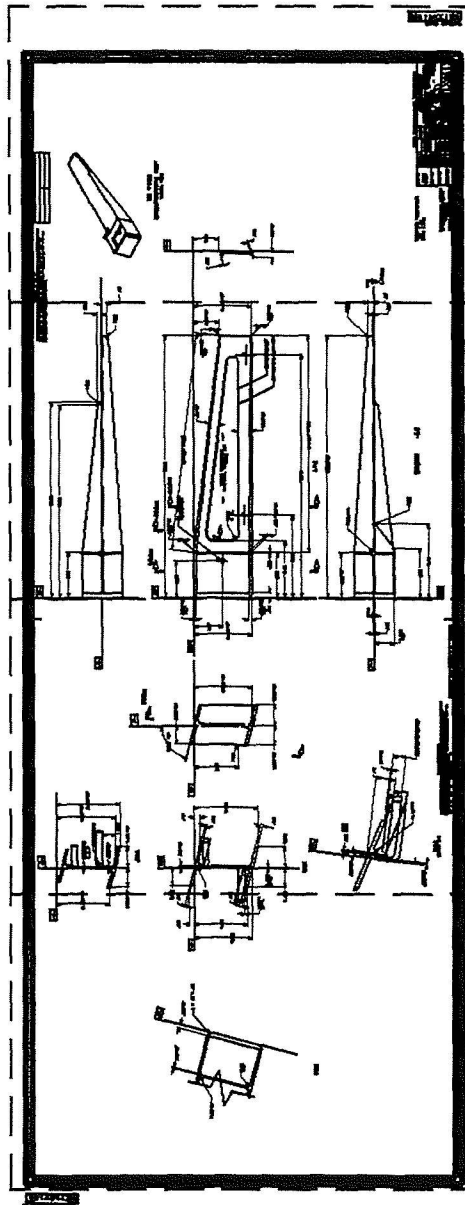
3D VIEW
ENGINEERING REF
SCALE: NONE

260

537 REF

DECIMALS TOLERANCES ±.XX ±.030 ±.XXX ±.010			
UNLESS OTHERWISE SPECIFIED DIMENSIONS ARE IN INCHES TOLERANCES: ANGLES ±.5° DECIMALS ±.001	TEST ONLY TEST NO. DATE NO. GROUP NO. 8-1845	DRAWN C. HENSON CHECKED DAVID B. ROLLAND DATE 12/7/93 12/26/93 12/7/93 12/7/93 12/7/93 12/7/93 12/7/93	BOEING COMPOSITE SERVICES SEATTLE, WA 98144 COMPOSITE SANDWICH SIDE PANEL-ASSY (TEST ONLY) J 8124 261X0413 SCALE 1/1 FOR 4

D1 8007 J440 REV 7/91
 261X0413



D-75

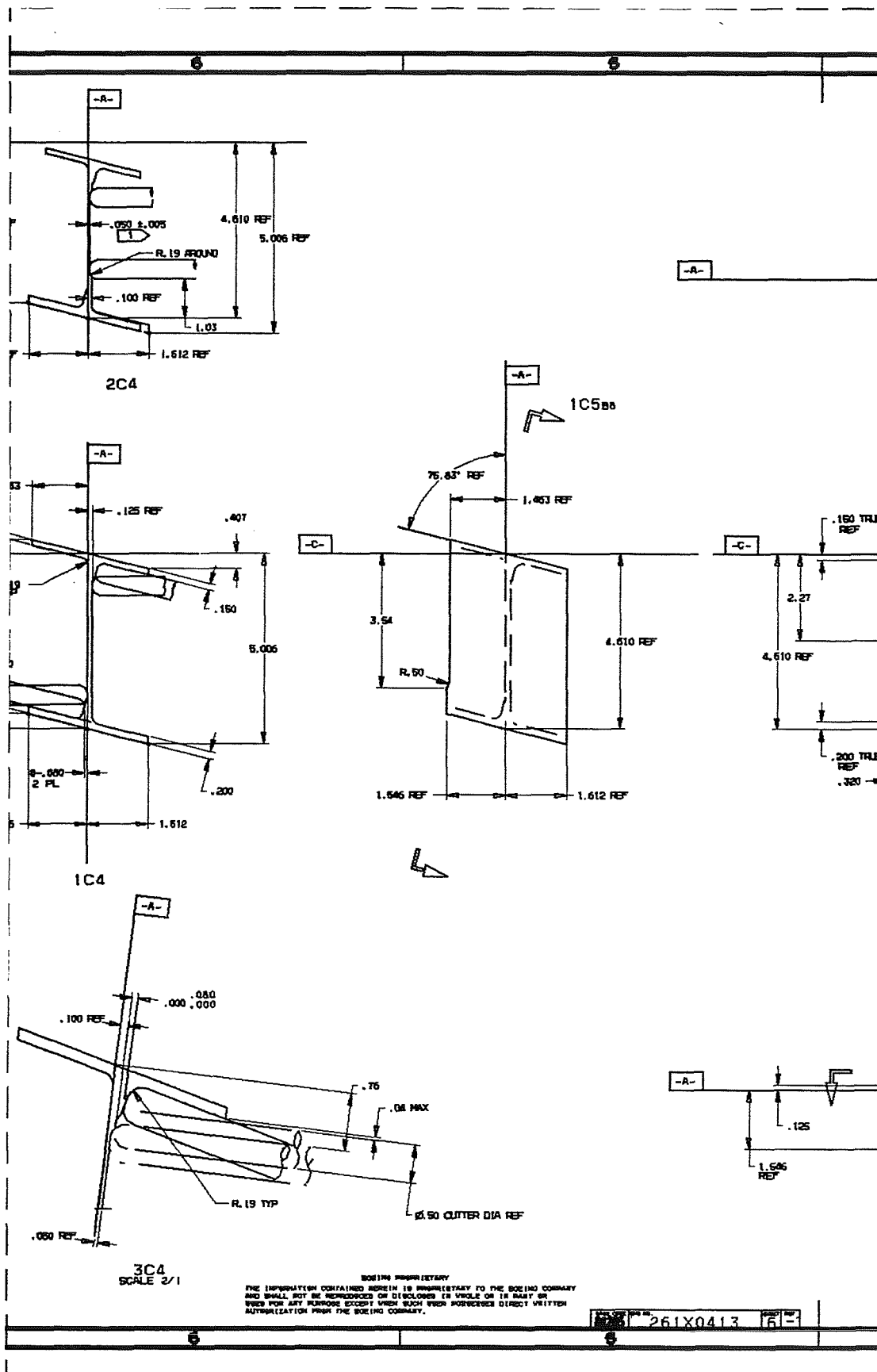
D-79

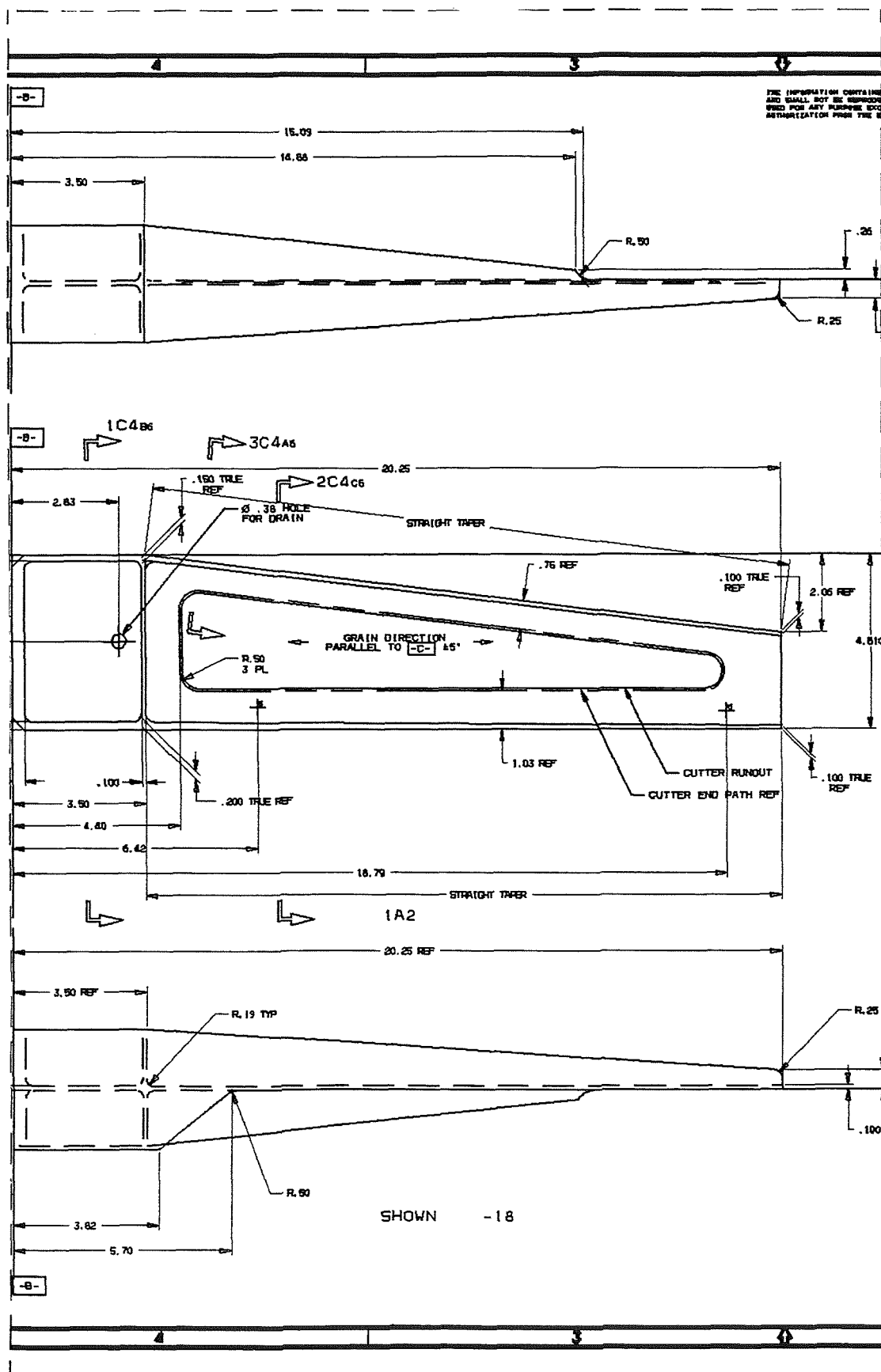
D-78

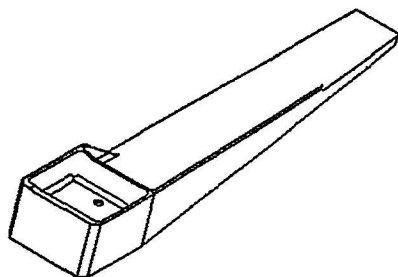
D-77

D-76

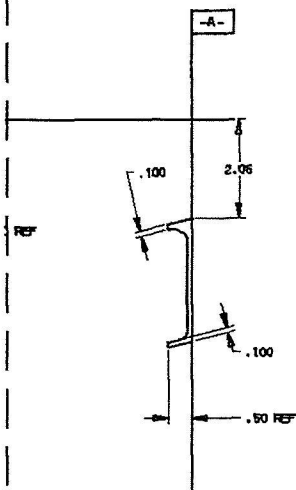








3D VIEW
ENGINEERING REF
SCALE: NONE



1A2B3

THE FOLLOWING INFORMATION SPECIFIED
 ON THIS FORM ARE IN THE
 RELEVANCE:
 ACCURACY: 2* DECIMALS: NOTE

SEE PL. FOR LIST OF MATERIAL, USAGE AND NOTES

VIDEO 4H	DISCO	BOEING
TEST ONLY	C. HARMON	(11/27/83)
	DAVID H. DOLLAND	(11/21/83)
TEST NO.	DAVID R. DOLLAND	(11/21/83)
	C. HARMON	(11/27/83)
VIDEO NO.	A. HENNINGSEN	(11/27/83)
	DAVID J. SMITH	(11/27/83)
VIDEO DISC	DAVID J. SMITH	(11/27/83)
	DAVID H. DOLLAND	(11/27/83)
R-YEAR	DAVID BOADLE	(12/7/83)

BOEING

CORPORATE OFFICES SEATTLE, WA 98144

COMPOSITE SANDWICH
SIDE PANEL - ASSY
(TEST ONLY)

VIDEO/CASS COPIES NO.

J 61005 261X0413

DISC #11 & NOTES

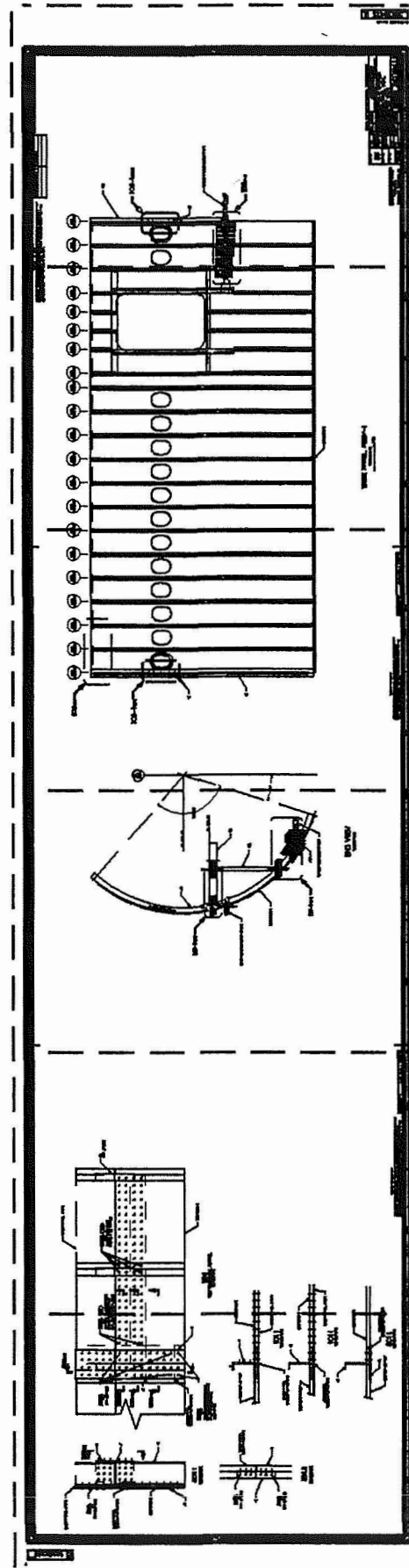
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D-82

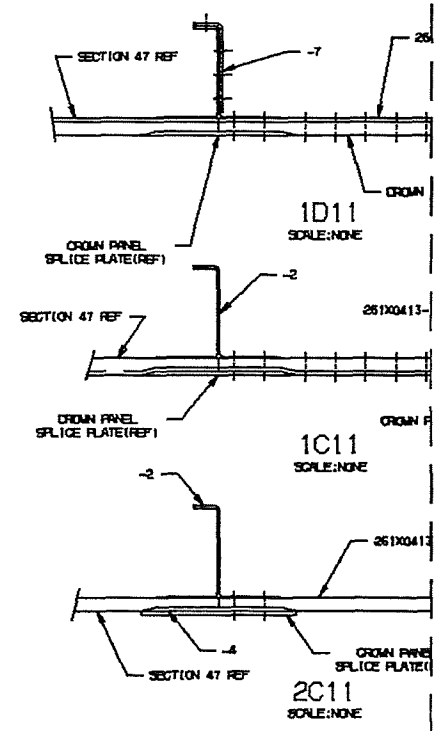
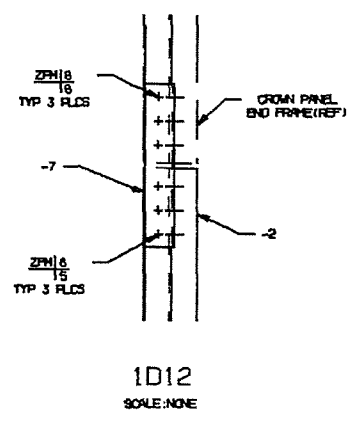
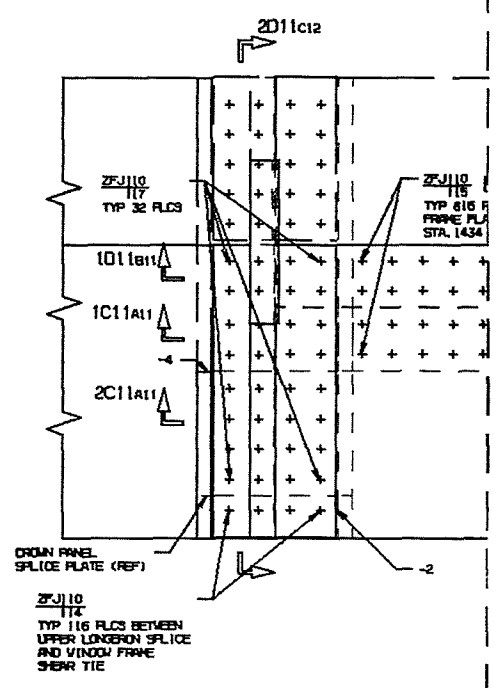
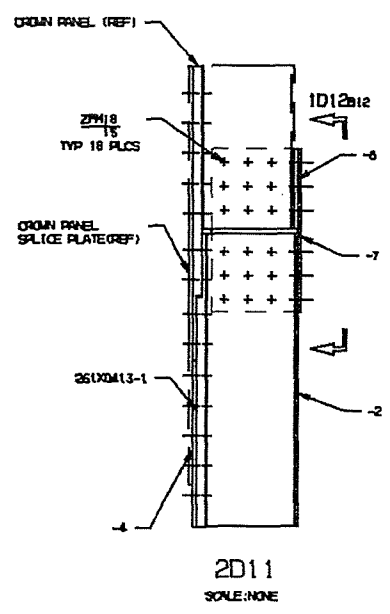
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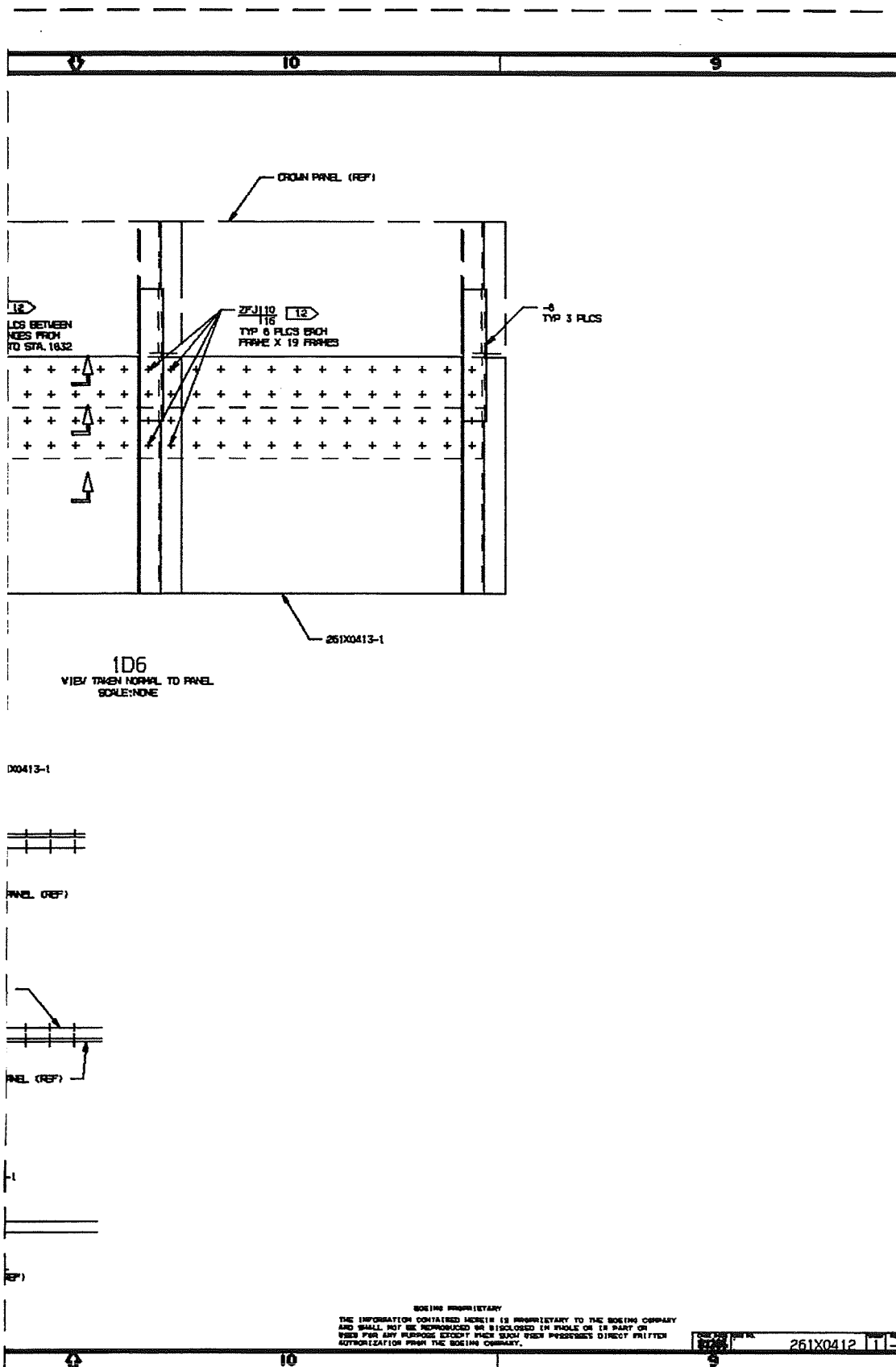
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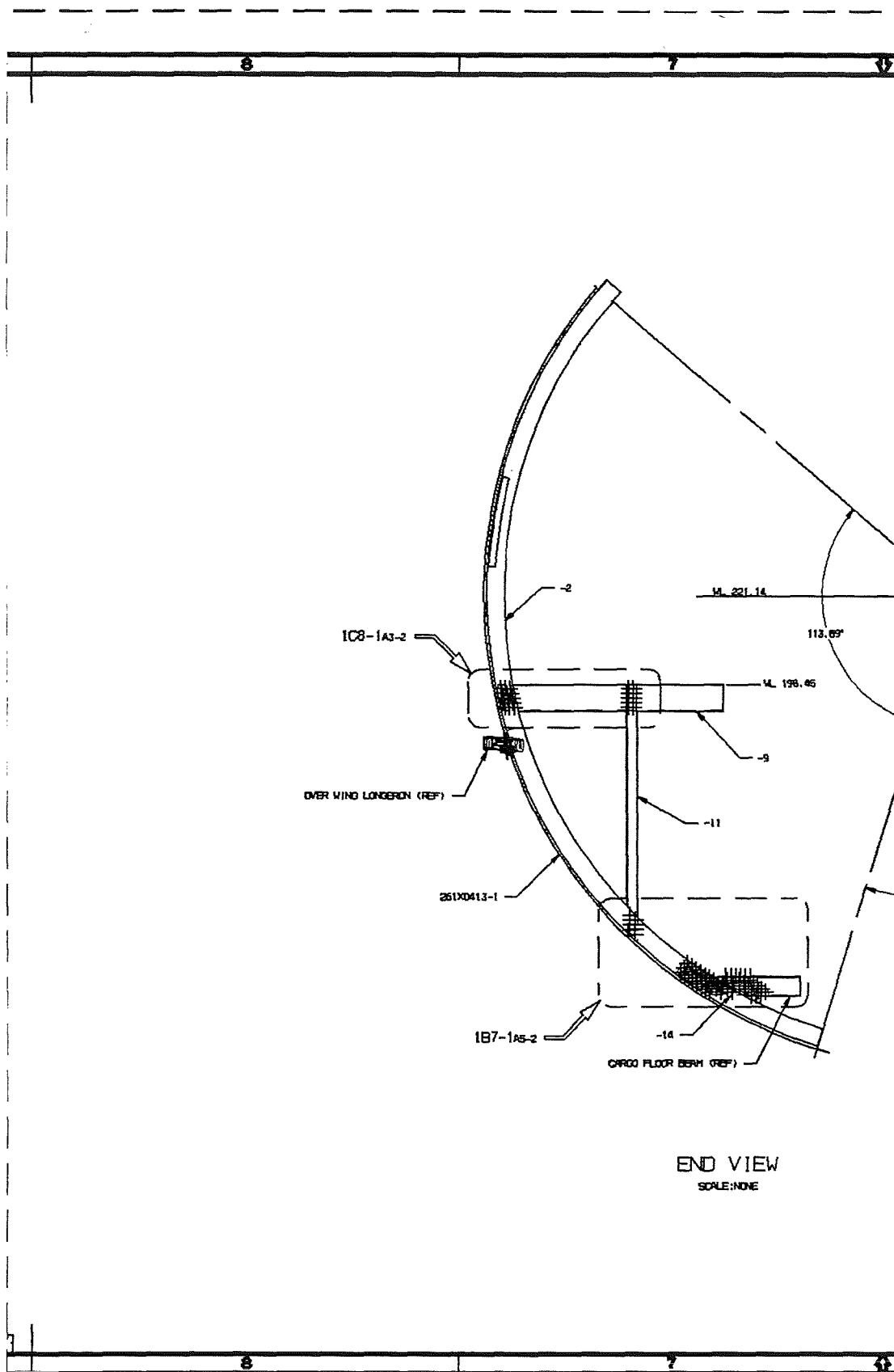
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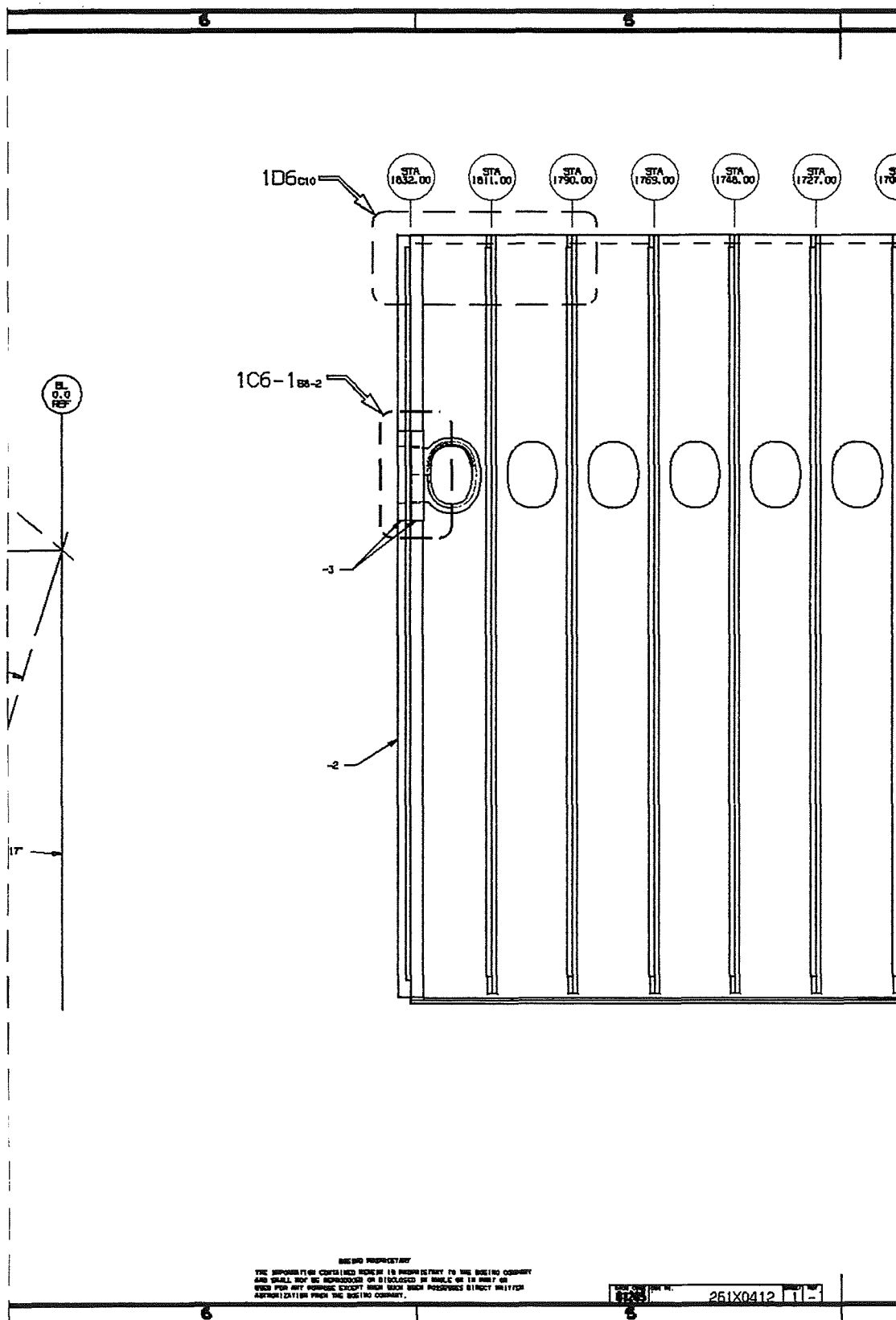
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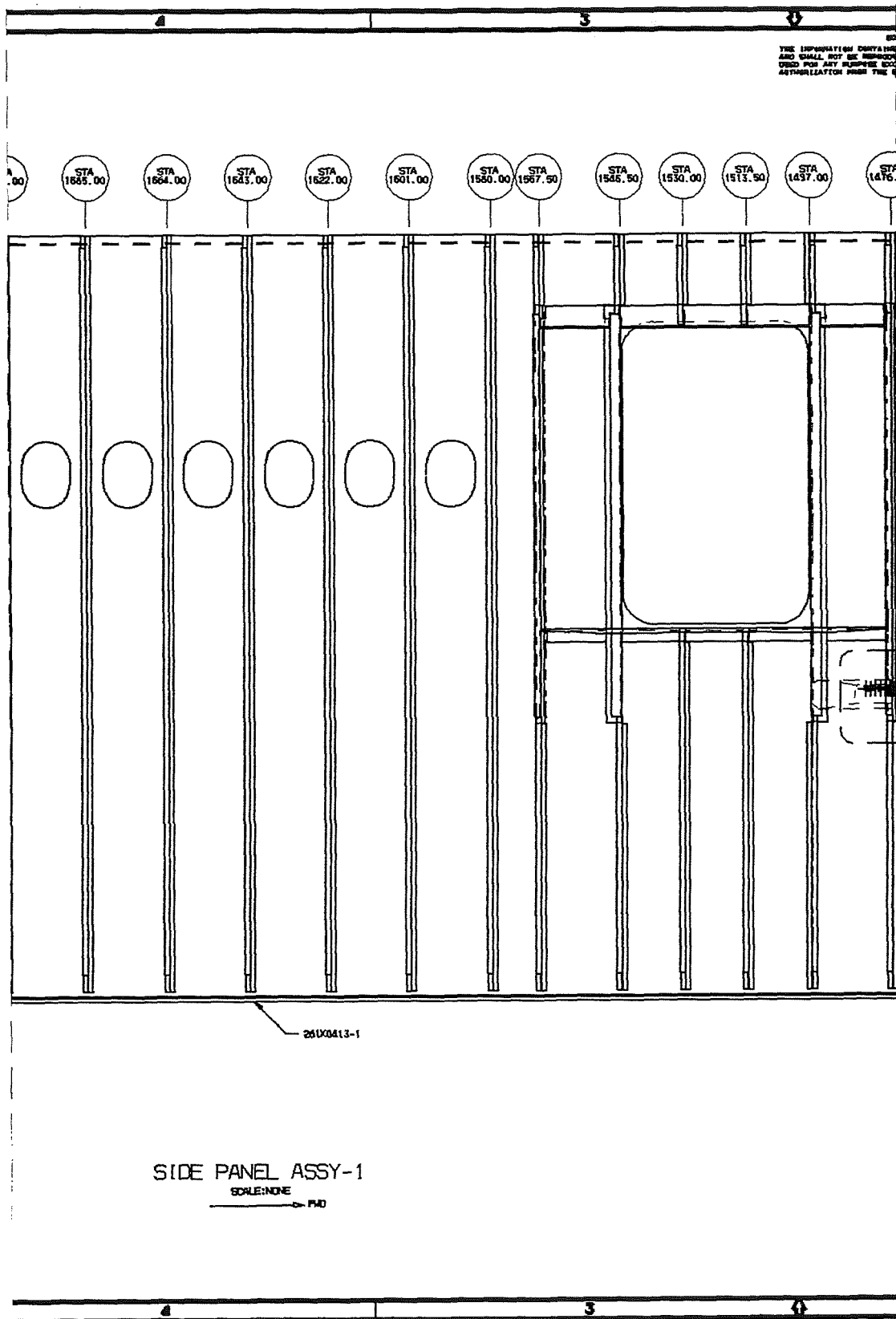
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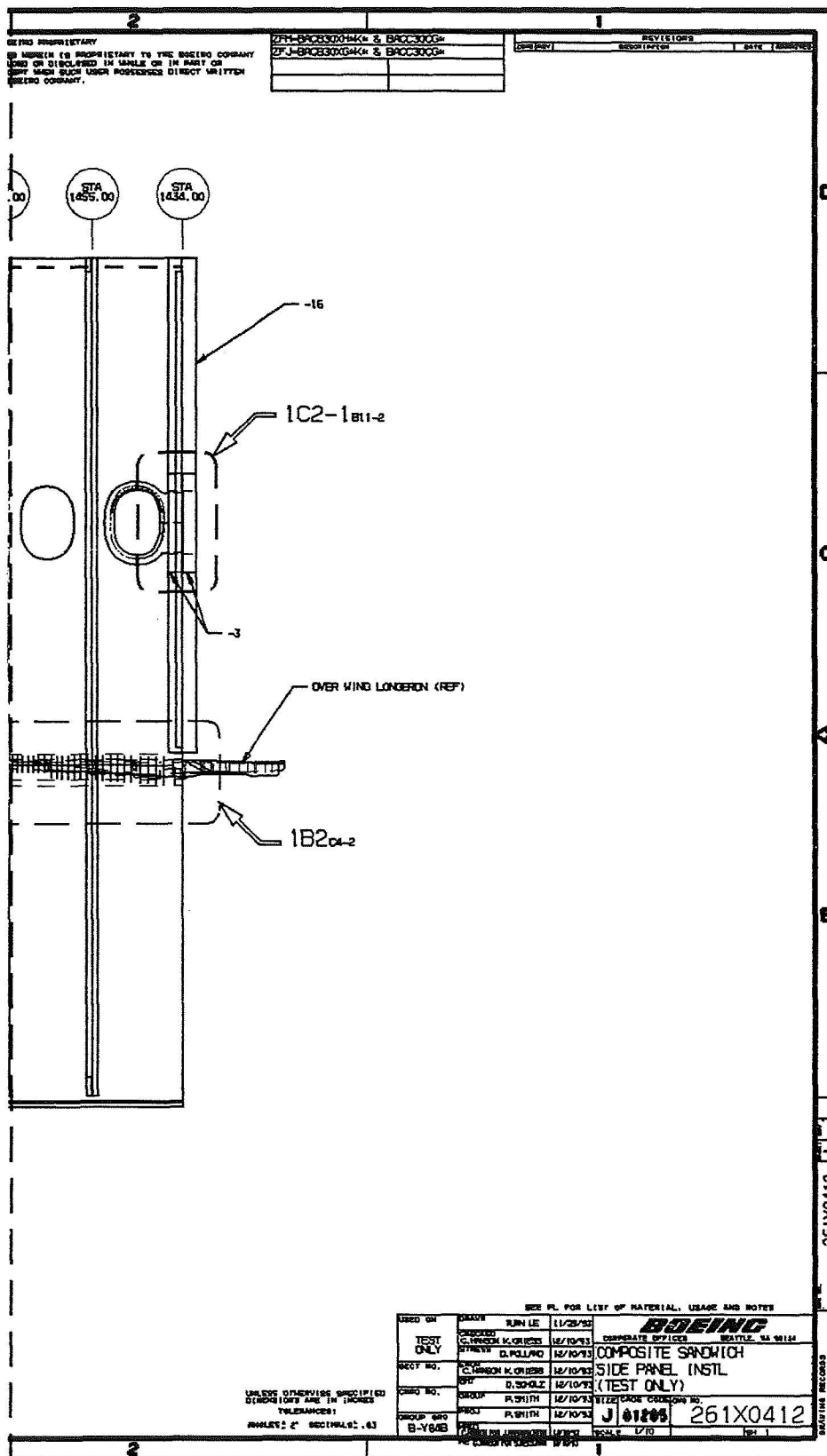






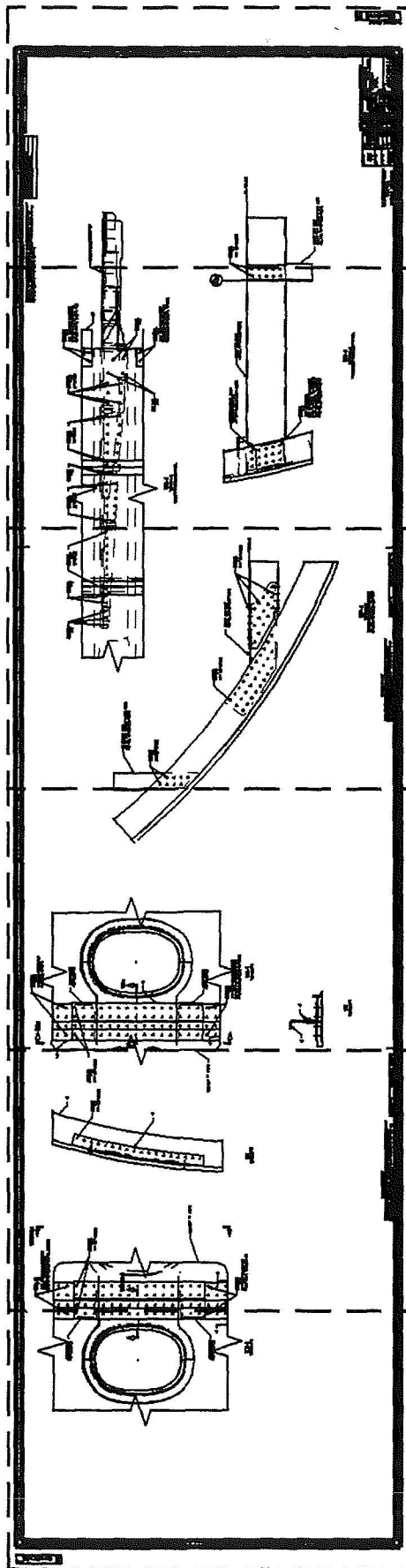






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D-88

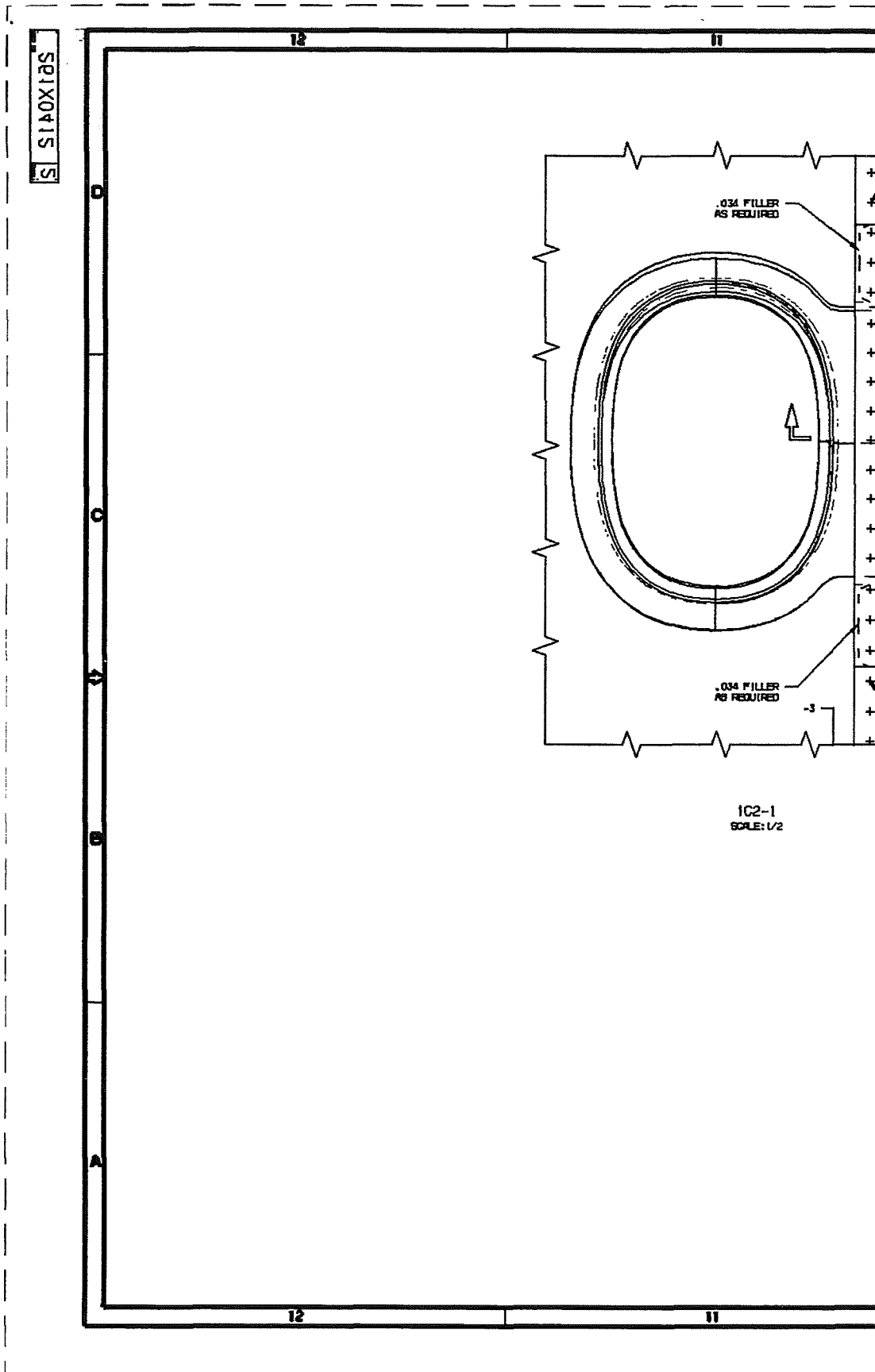
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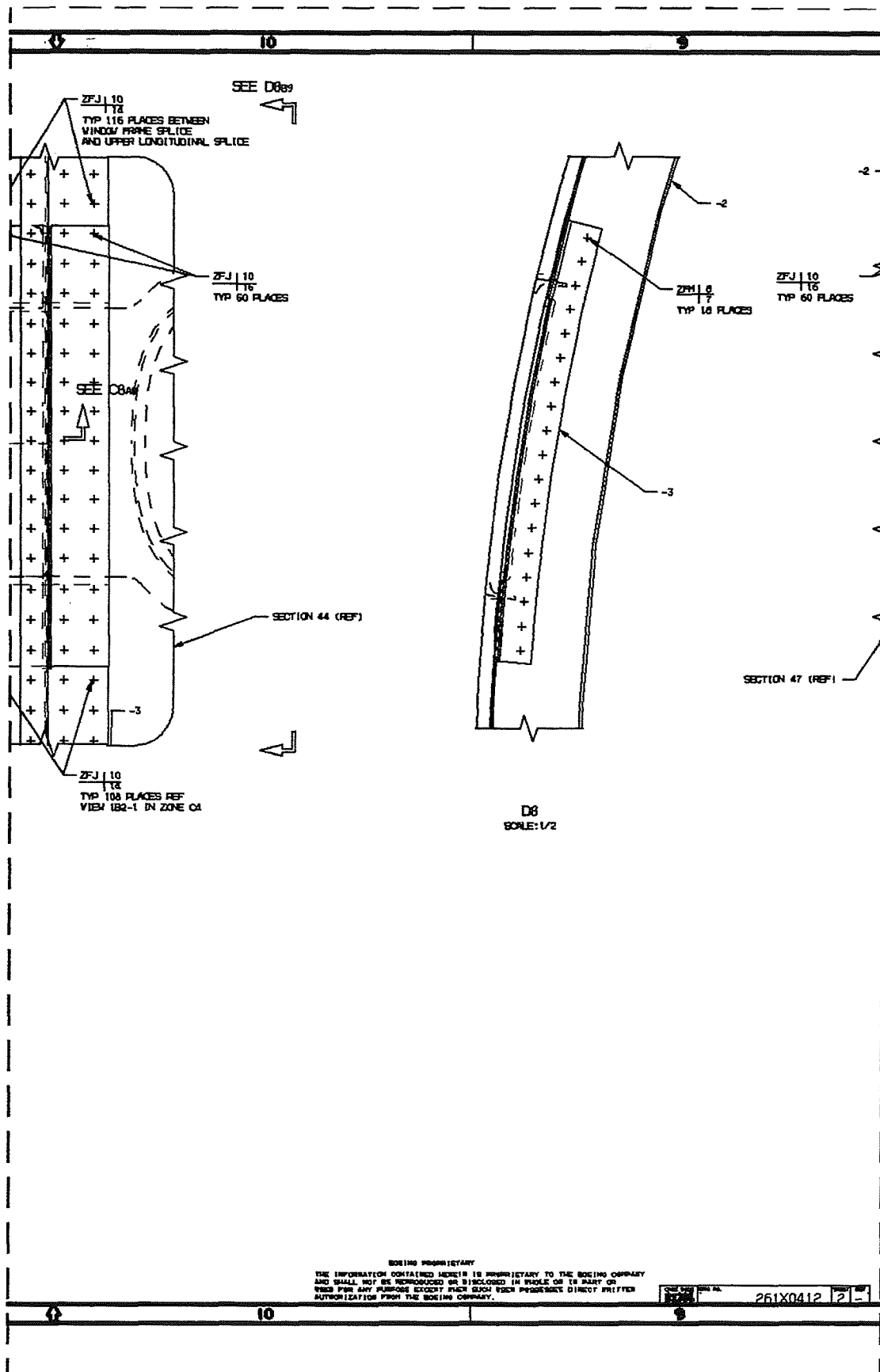
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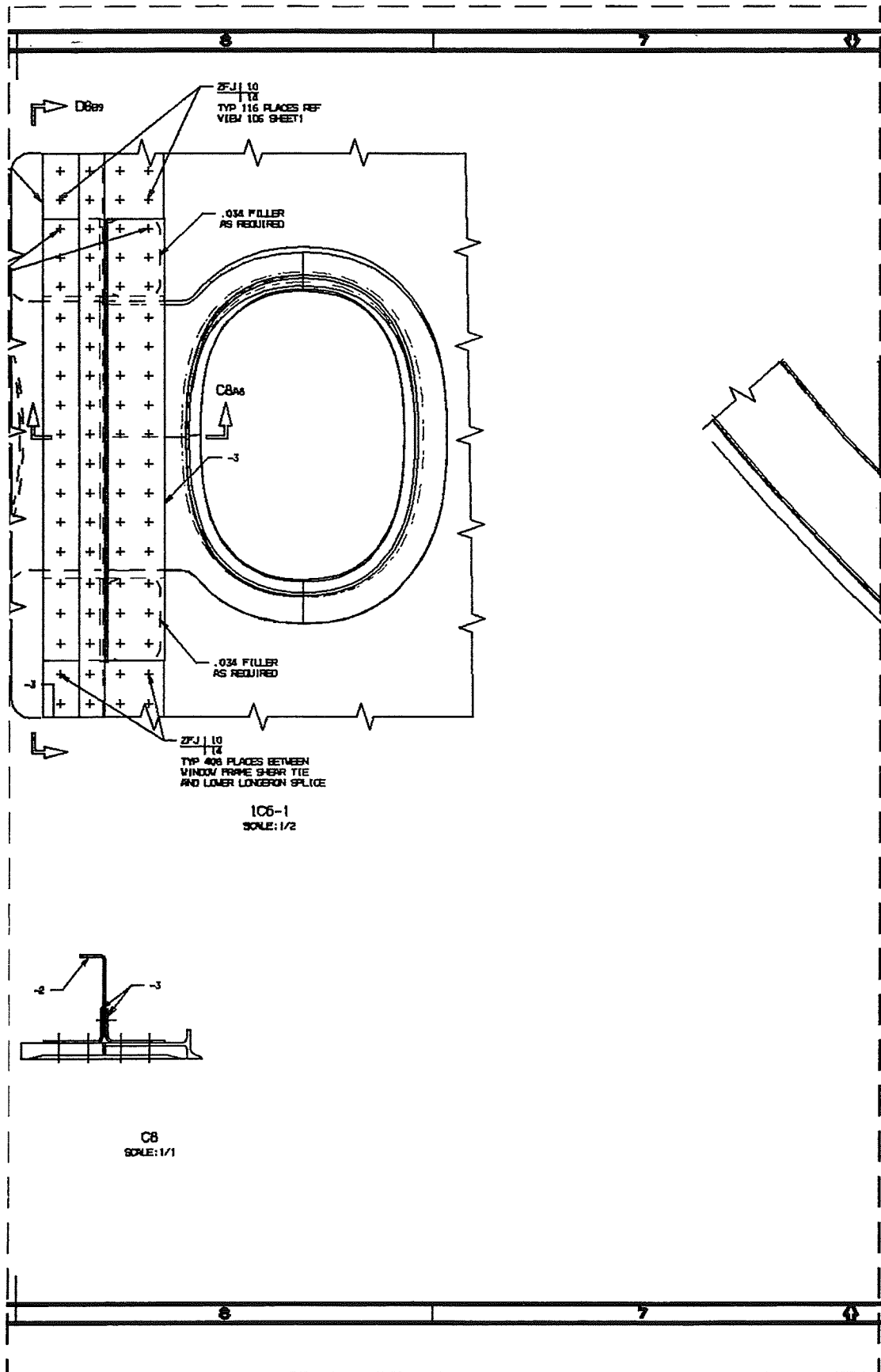
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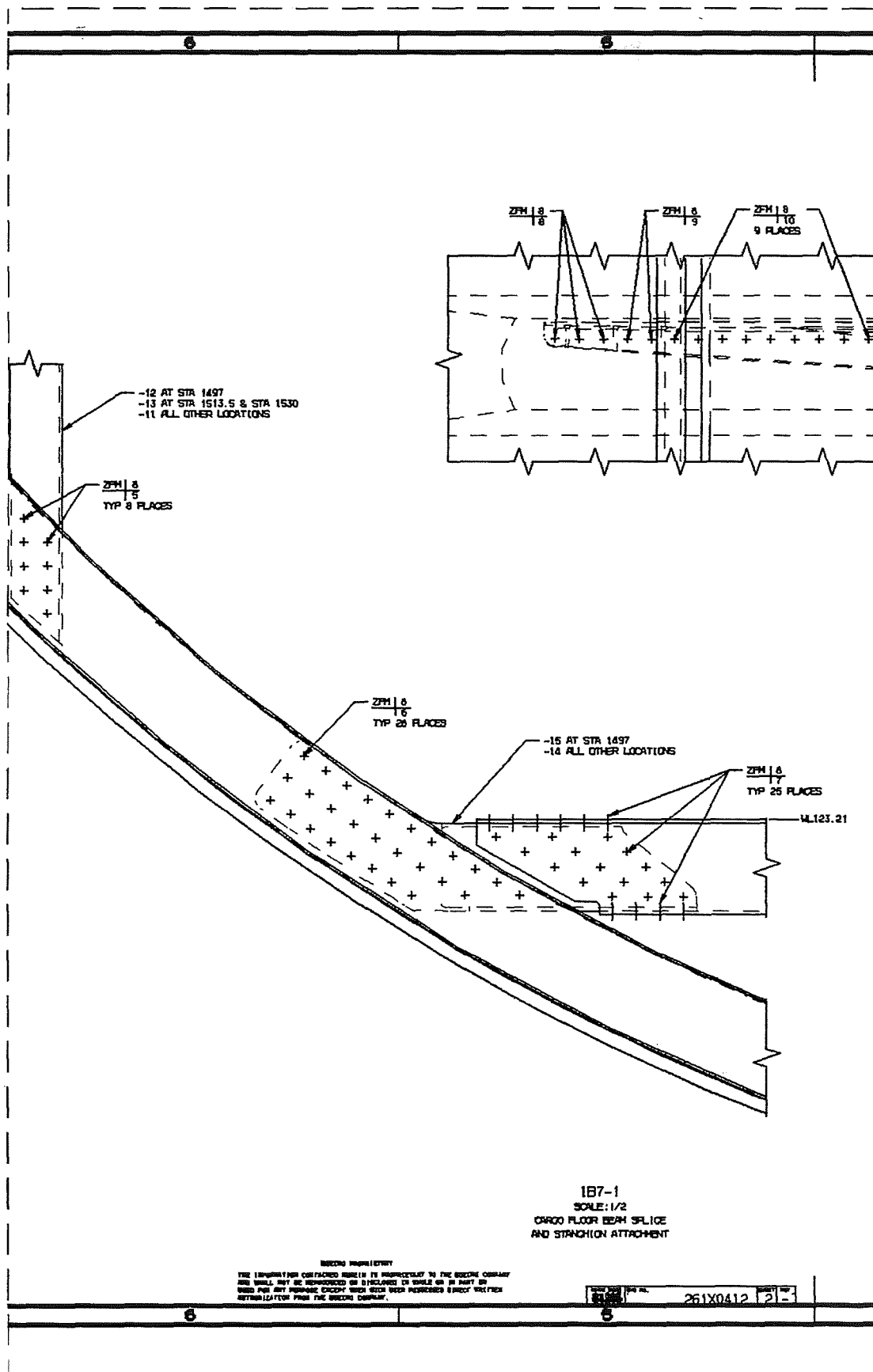
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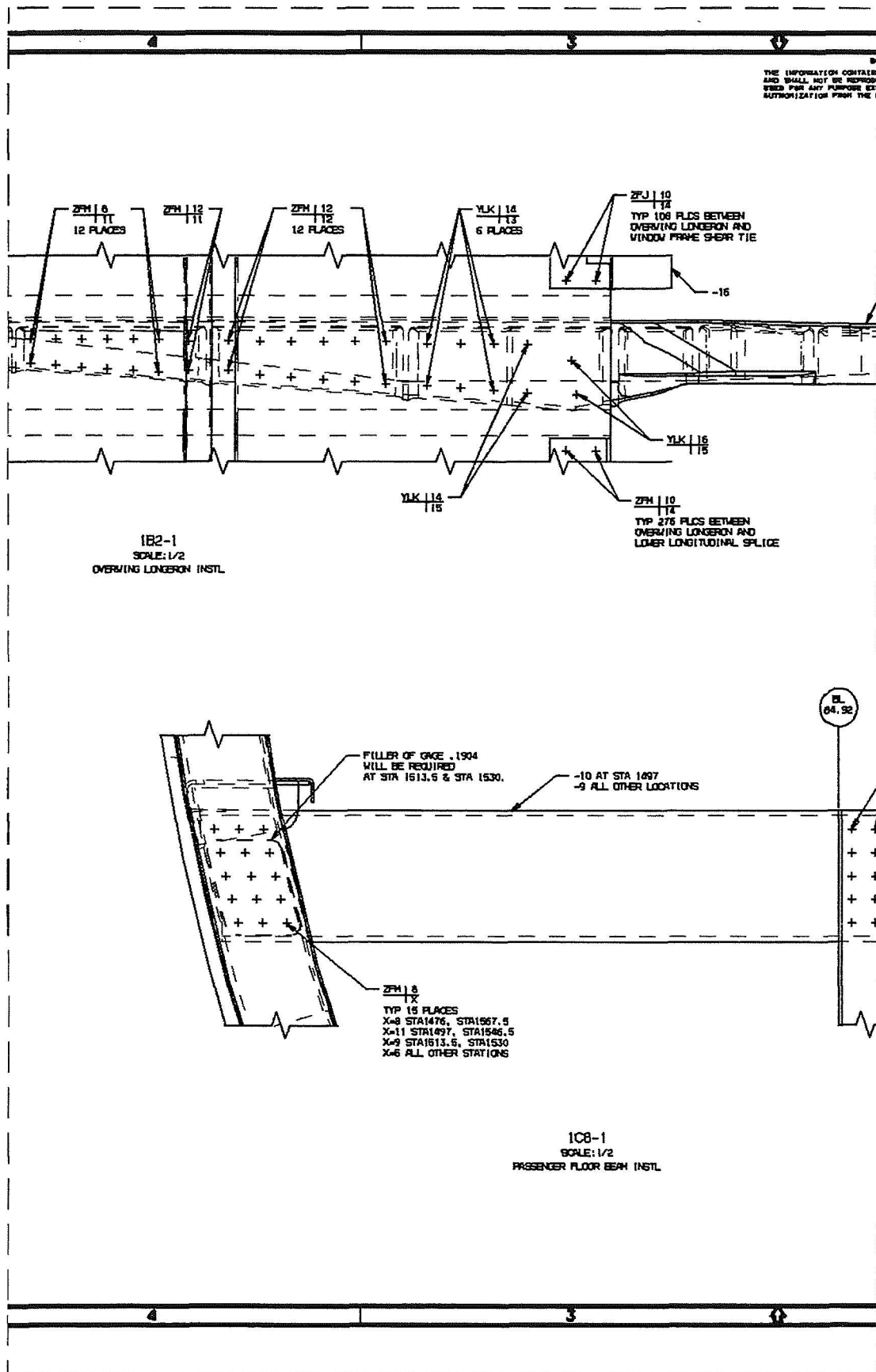
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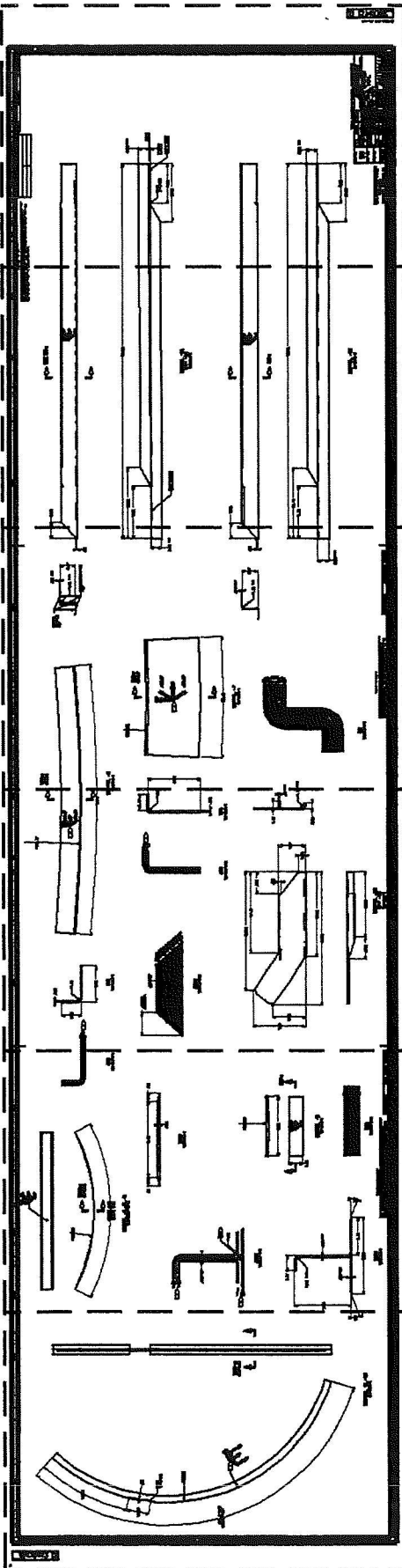












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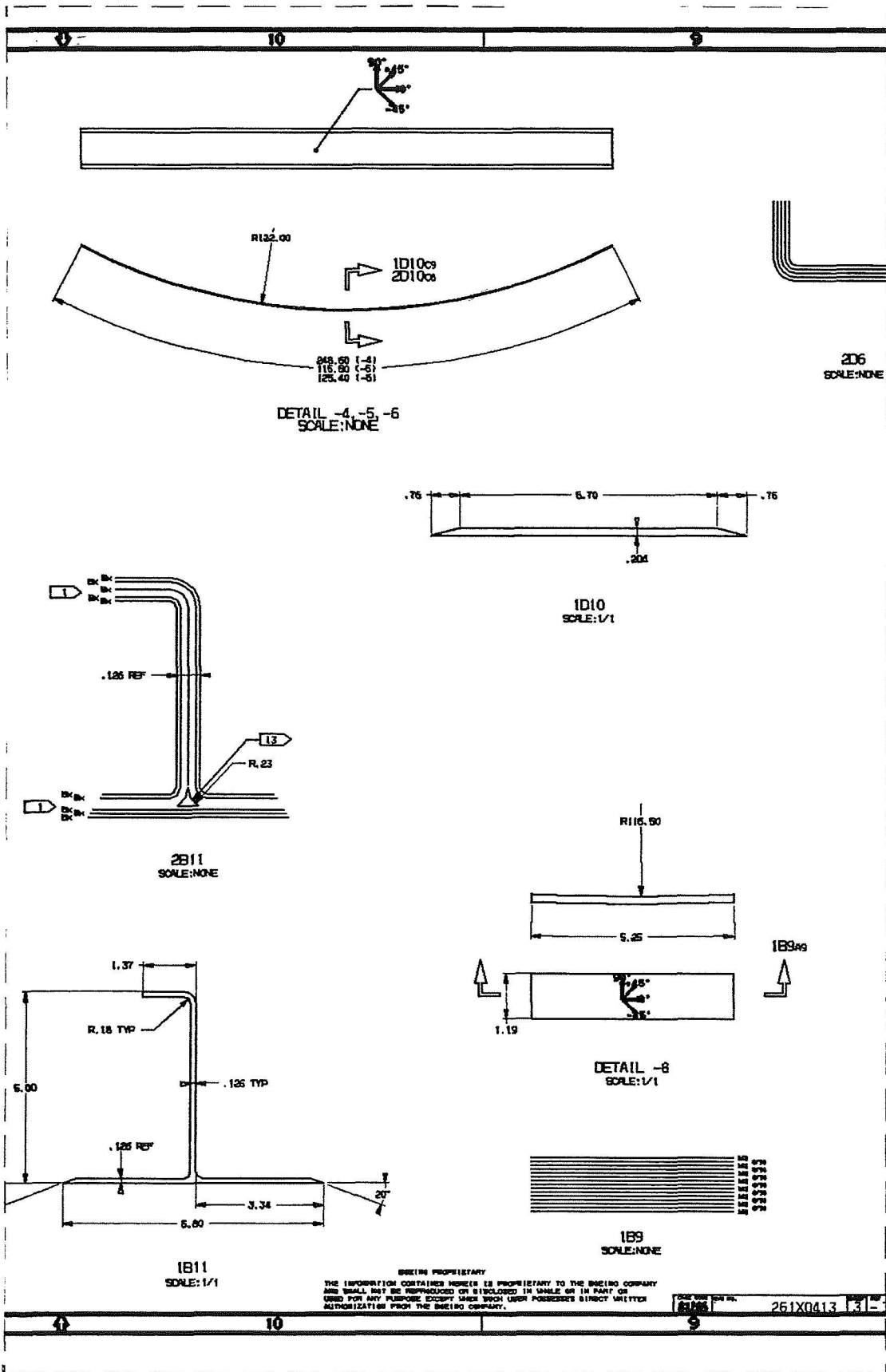
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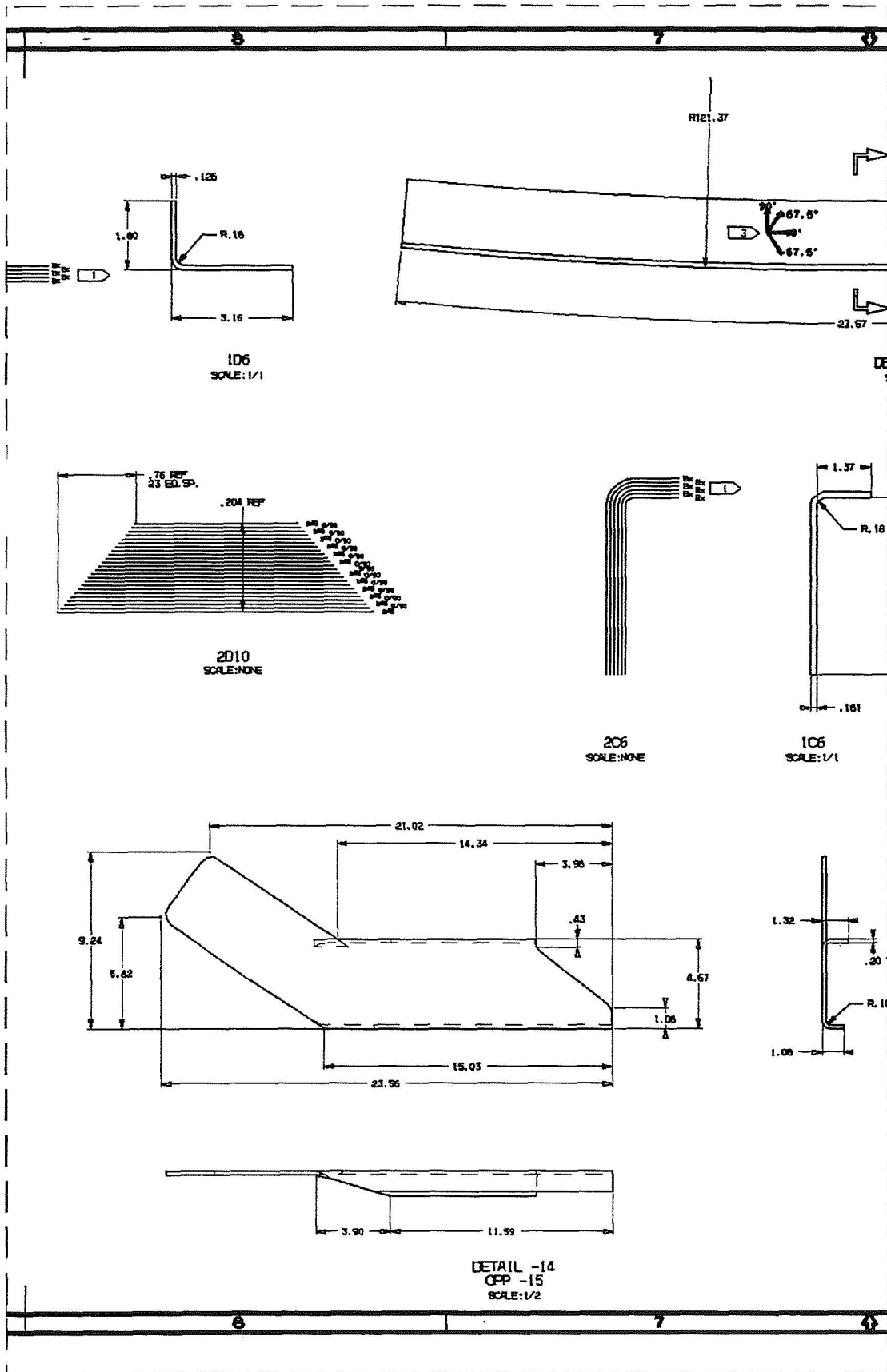
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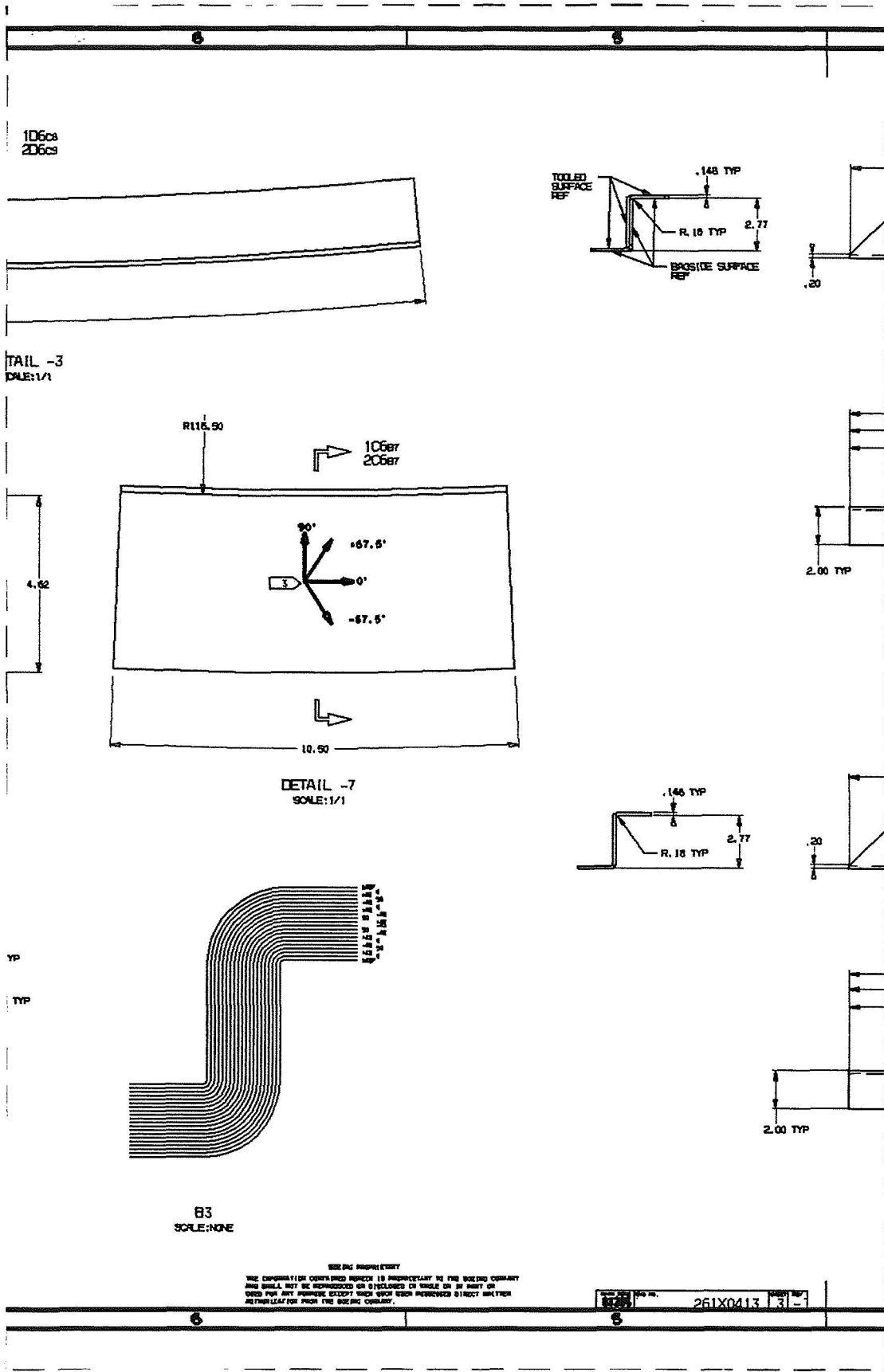
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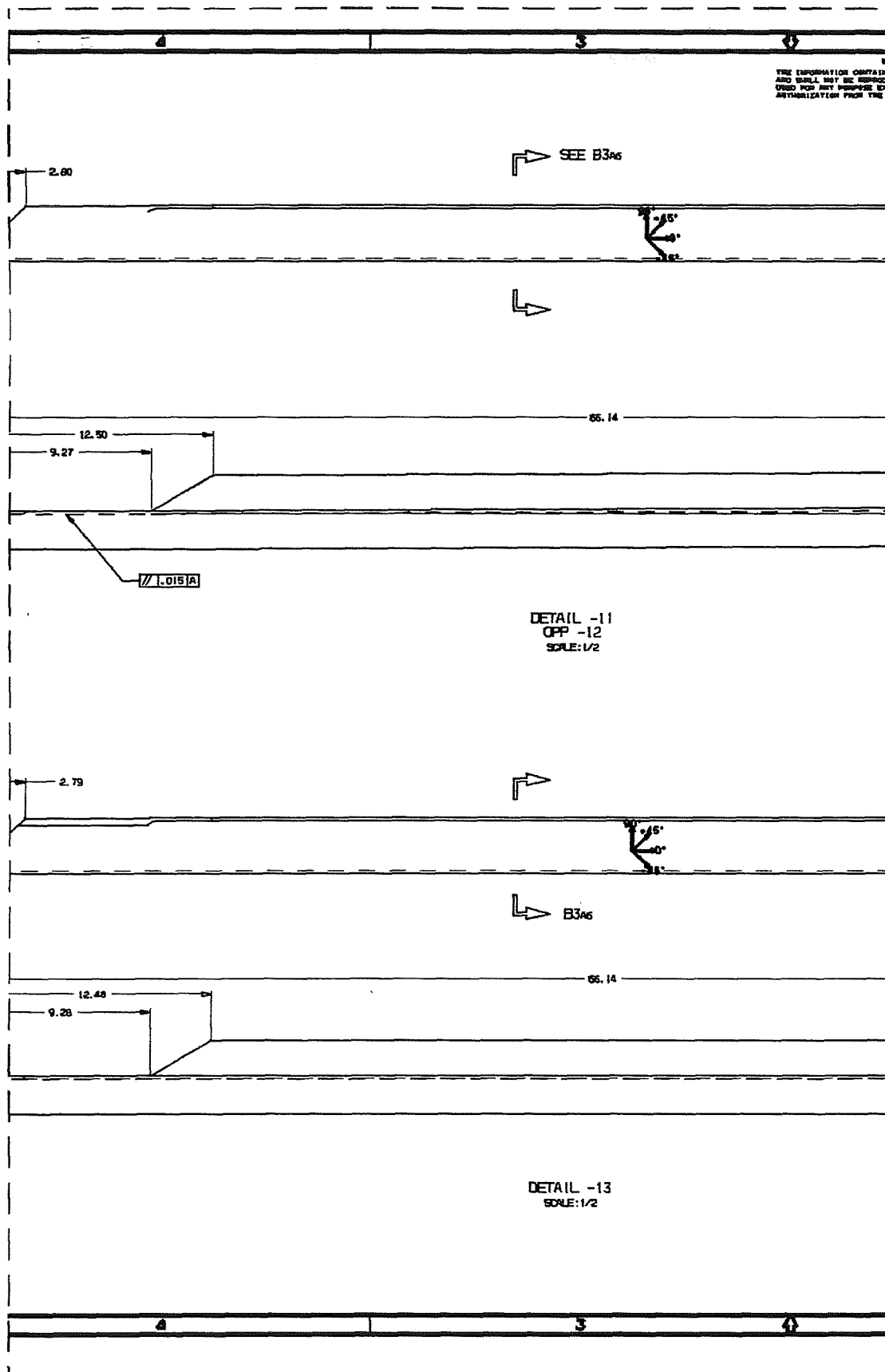
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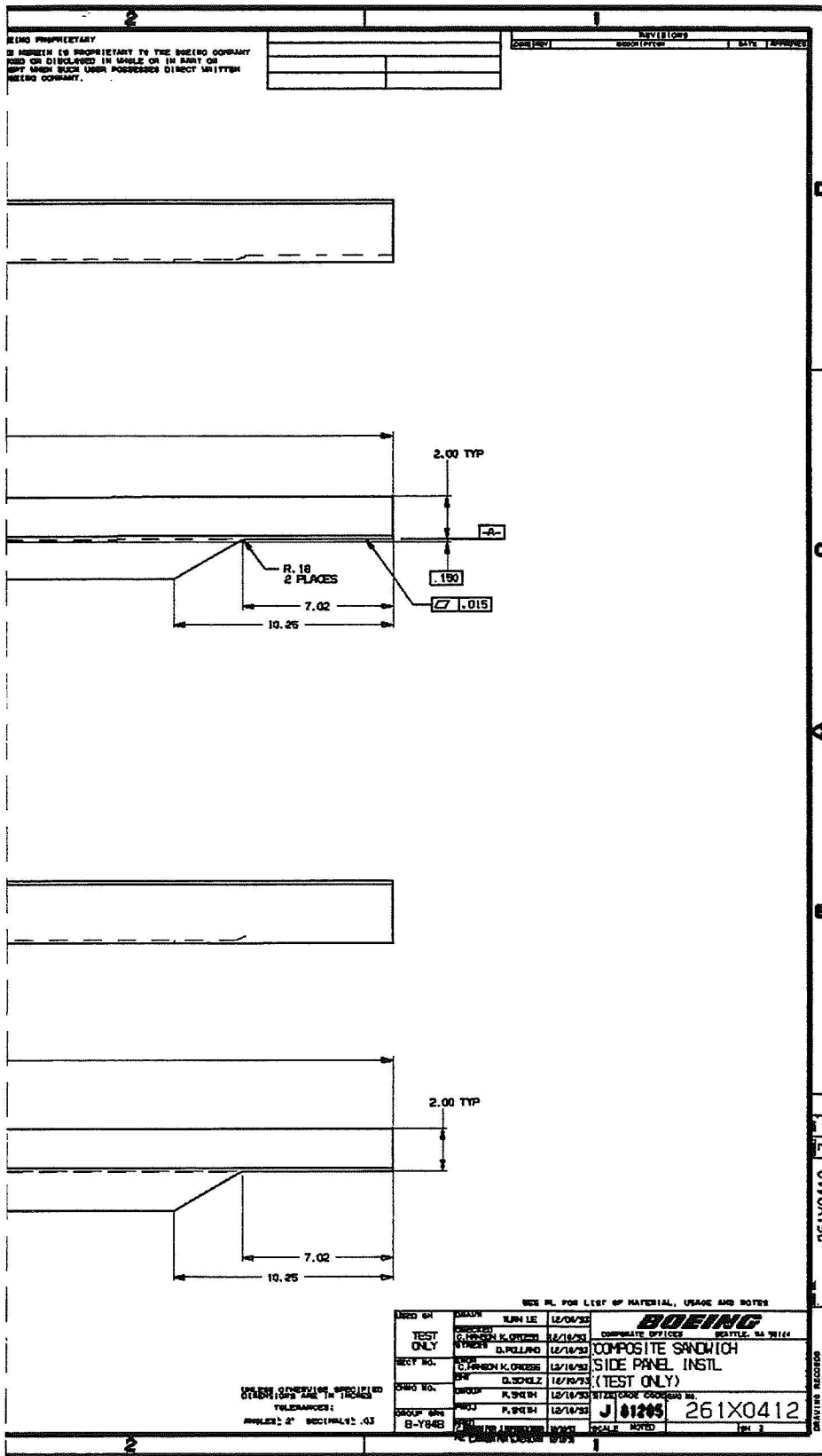
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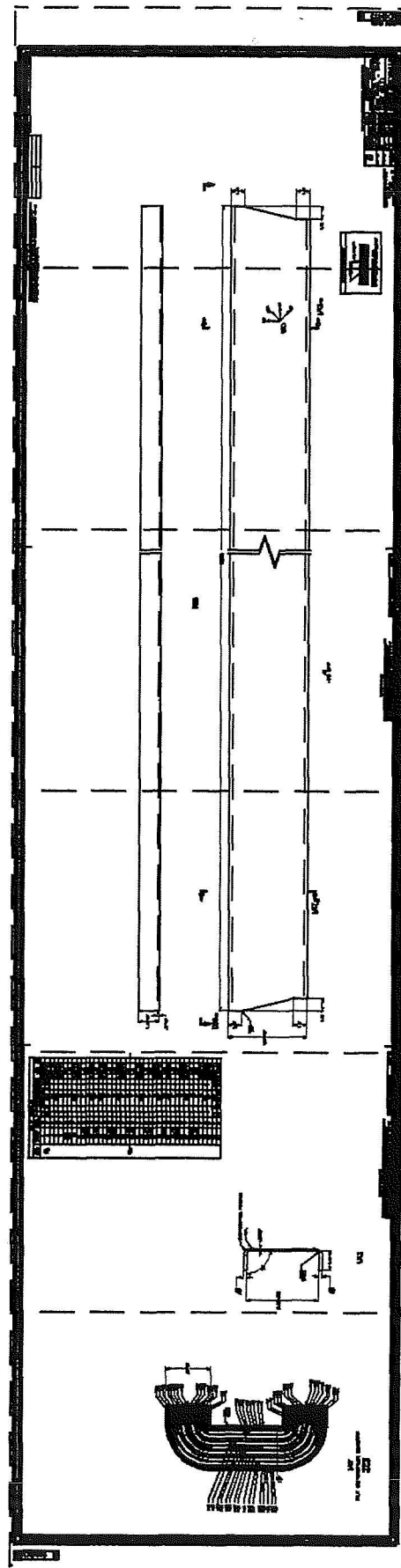




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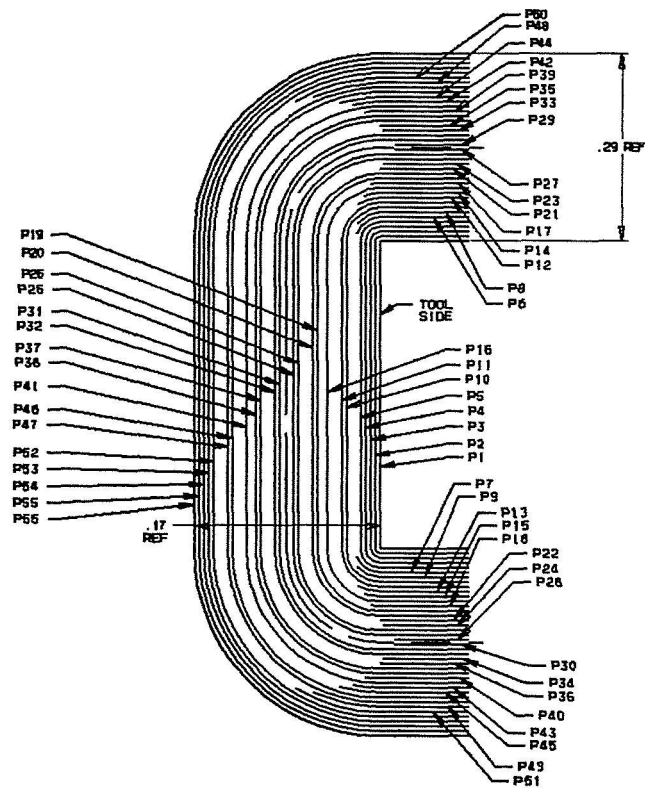
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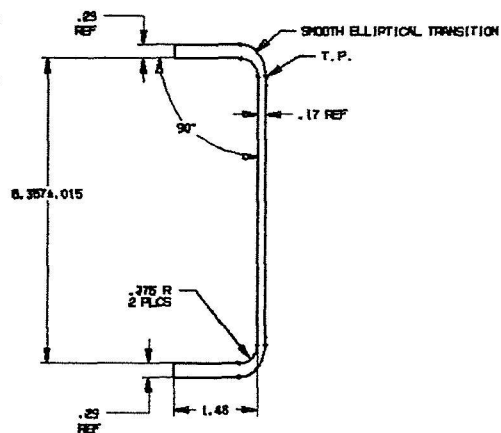
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PLY DEFINITION DIAGRAM
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REF ONLY

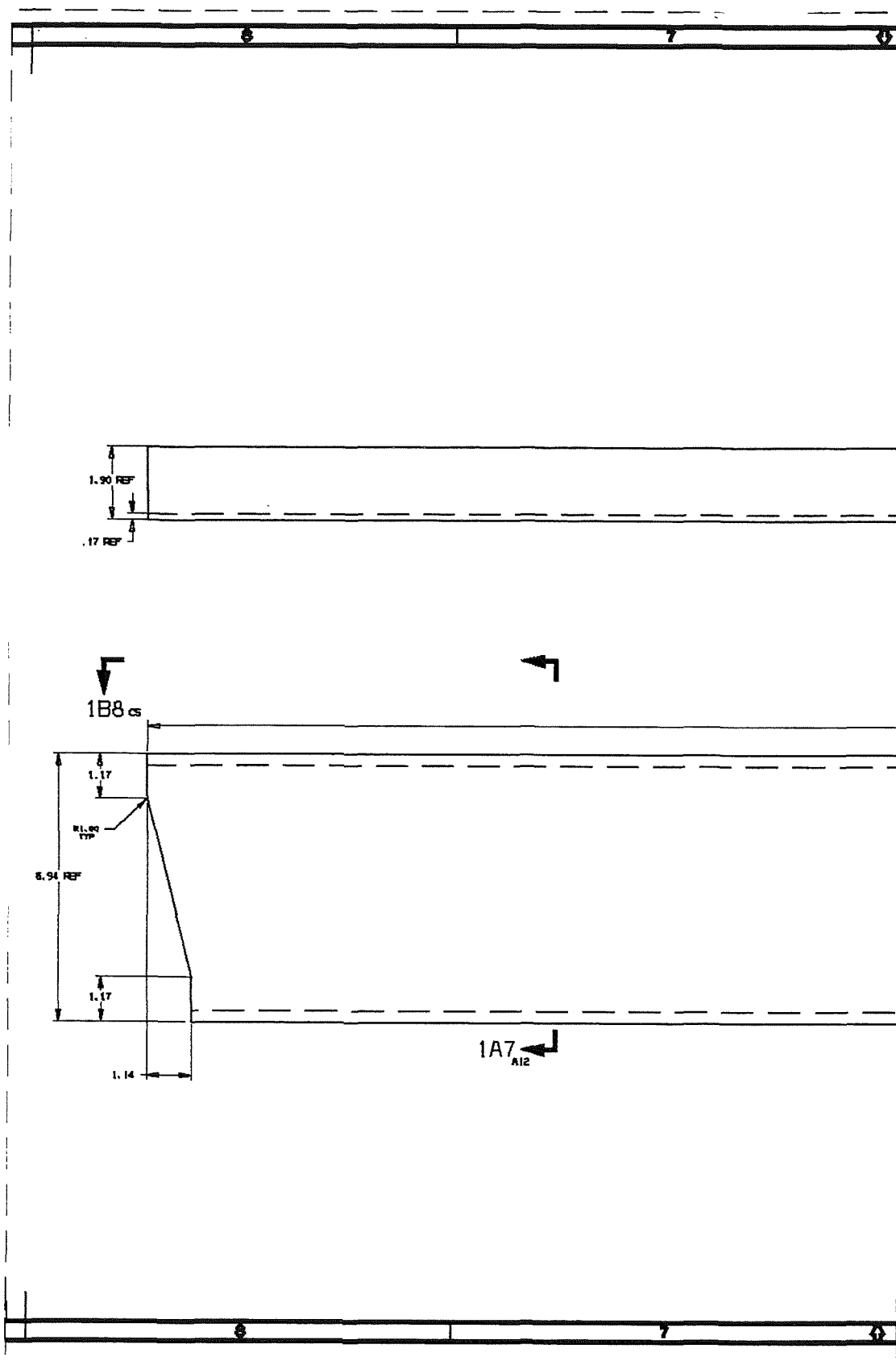
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-9.	1	F	9	0° OR 90°	10	9.80 MIN
-10	2	F	8	+45°	10	9.83 MIN
	3	F		90°		9.86 MIN
	4	F		-45°		9.89 MIN
	5	F		0°		9.92 MIN
	6, 7	P		0°		1.87
	8, 9	P		0°		1.82
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	11	F		0°		9.89 MIN
	12, 13	P		0°		1.75
	14, 15	P		0°		1.70
	16	F		-45°		9.72 MIN
	17, 18	P		90°		1.67
	19	F		+45°		9.76 MIN
	20	F		0°		9.78 MIN
	21, 22	P		0°		1.63
	23, 24	P		0°		1.56
	25	F		-45°		9.61 MIN
	26	F		0°		9.64 MIN
	27, 28	P		0°		1.51
	29, 30	P		0°		1.51
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	35, 36	P		0°		1.70
	37	F		0°		9.93 MIN
	38	F		+45°		9.97 MIN
	39, 40	P		90°		1.75
	41	F		-45°		9.99 MIN
	42, 43	P		0°		1.82
	44, 45	P		0°		1.87
	46	F		0°		10.03 MIN
	47	F		+45°		10.06 MIN
	48, 49	P		0°		1.99
	50, 51	P		0°		2.06
	52	F		0°		10.09 MIN
	53	F		-45°		10.12 MIN
	54	F		90°		10.15 MIN
	55	F	8	+45°	10	10.18 MIN
	56	F	9	0° OR 90°	10	10.21 MIN

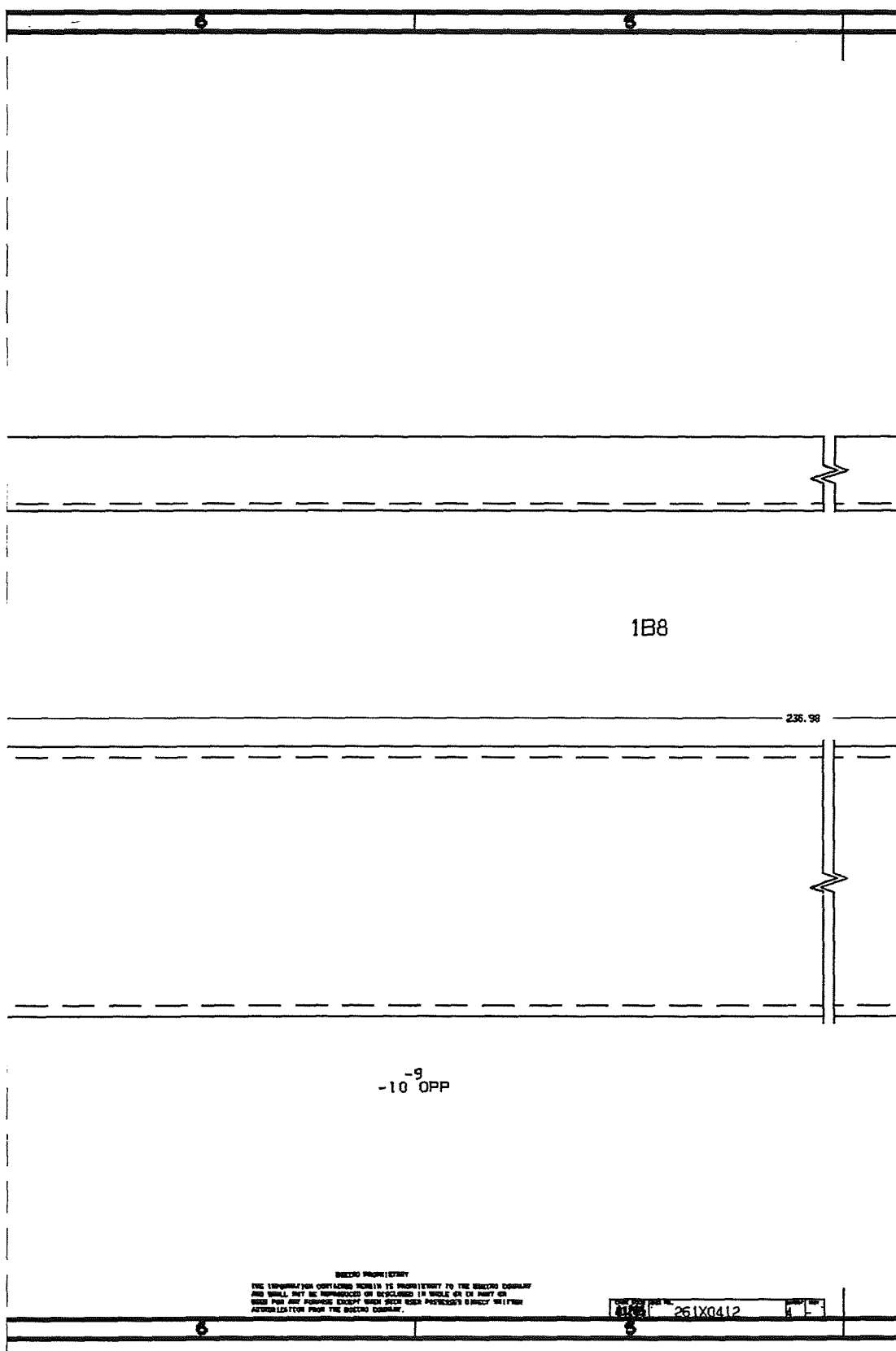


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PLY DEF

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REPORT DOCUMENTATION PAGE			Form Approved OMB No. 0704-0188	
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13. ABSTRACT (Maximum 200 words) <p>This report documents preliminary design trades conducted under NASA contracts NAS1 18889 (Advanced Technology Composite Aircraft Structures, ATCAS) and NAS1-19349 (Task 3, Pathfinder Shell Design) for a subsonic wide body commercial aircraft fuselage side panel section utilizing composite materials.</p> <p>Included in this effort were (a) development of two complete design concepts, (b) generation of cost and weight estimates, (c) identification of technical issues and potential design enhancements, and (d) selection of a single design to be further developed. The first design concept featured an open-section stringer stiffened skin configuration while the second was based on honeycomb core sandwich construction.</p> <p>The trade study cost and weight results were generated from comprehensive assessment of each structural component comprising the fuselage side panel section from detail fabrication through airplane final assembly. Results were obtained in three phases: (1) for the baseline designs, (2) after global optimization of the designs, and (3) the results anticipated after detailed design optimization. A critical assessment of both designs was performed to determine the risk associated with each concept, that is the relative probability of achieving the cost and weight projections. Seven critical technical issues were identified as the first step towards side panel detailed design optimization.</p>				
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