



Solar Stirling for Deep Space Applications

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Prepared for the
Space Technology and Applications International Forum—2000
sponsored by the American Institute of Physics
Albuquerque, New Mexico, January 30—February 3, 2000

National Aeronautics and
Space Administration

Glenn Research Center

This report contains preliminary findings, subject to revision as analysis proceeds.

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Abstract. A study was performed to quantify the performance of solar thermal power systems for deep space planetary missions. The study incorporated projected advances in solar concentrator and energy conversion technologies. These technologies included inflatable structures, lightweight primary concentrators, high efficiency secondary concentrators, and high efficiency Stirling convertors. Analyses were performed to determine the mass and deployed area of multi-hundred watt solar thermal power systems for missions out to 40 astronomical units. Emphasis was given to system optimization, parametric sensitivity analyses, and concentrator configuration comparisons. The results indicated that solar thermal power systems are a competitive alternative to radioisotope systems out to 10 astronomical units without the cost or safety implications associated with nuclear sources.

INTRODUCTION

The traditional means to satisfy electrical power requirements for outer planetary space probes is through Radioisotope Thermoelectric Generators (RTGs). RTGs were most recently used on the Galileo, Ulysses and Cassini spacecraft (Kelly, 1997). A joint DOE/NASA program is in place to develop an improved radioisotope power system to replace RTGs. The higher efficiency, Advanced Radioisotope Power System (ARPS) will reduce the required plutonium inventory providing cost and safety benefits (Herrera, 1998). ARPS will utilize an Alkali-Metal Thermal-to-Electric Converter (AMTEC) combined with three General Purpose Heat Source (GPHS) modules to produce 92 watts beginning-of-mission (BOM) in a single power unit. The AMTEC system is projected to offer at least a 2x improvement in conversion efficiency as compared to conventional thermoelectric converters used in RTGs. This technology is planned for use on future deep space science missions such as Europa Orbiter ('03) and Pluto-Kuiper Express ('04). The projected specific power for the ARPS system is 6.2 W/kg and the efficiency is 12.6% at BOM (Lockheed Martin, 1998)

While reducing the amount of plutonium reduces the health risk associated with an accidental orbital reentry and provides substantial system cost savings, it would be desirable to have a non-nuclear option for deep space missions. However, typical planar photovoltaic (PV) arrays are not effective for space probes traveling beyond Mars (1.5 astronomical units, or AU) due to the decrease in insolation with the square of the distance from the sun.



FIGURE 1. Lightweight Inflatable Concentrator (courtesy of SRS Technologies)

Solar thermal power systems offer a potential alternative. Progress in advanced lightweight concentrator technology provides a necessary first step toward making solar thermal power for deep space missions a viable option. Companies such as L'Garde, SRS Technologies, ILC Dover, United Applied Technologies, and Harris Corporation are developing concepts for large, lightweight solar concentrators. Figure 1 shows an example of a lightweight concentrator using thin-film, inflatable technology. This advanced concentrator technology offers a factor of five improvement in aerial density (kg/m^3) over conventional rigid panel concentrators (Mason, 1999). The other key elements to a mass competitive solar power system for far-sun missions are high efficiency secondary concentrators and high efficiency, free-piston Stirling convertors.

Secondary concentrators can provide an increase in the overall geometric concentration ratio as compared to primary concentrators alone. This reduces the diameter of the receiver aperture and the associated infrared cavity losses, thus improving overall efficiency. The use of a secondary concentrator also eases the pointing and surface accuracy requirements of the primary concentrator, making inflatable structures a more feasible option. Typical secondary concentrators are hollow, reflective parabolic cones. Recent studies at Glenn Research Center have investigated the use of a solid, crystalline refractive secondary concentrator for solar thermal propulsion which may provide considerable improvement in throughput efficiency by eliminating reflective losses (Wong, 1999). The refractive secondary concept, shown in Figure 2, also offers the benefit of directed flux tailoring within the receiver cavity via a unique "flux extractor." Such a device has the potential to improve the energy transfer to the Stirling heater head.

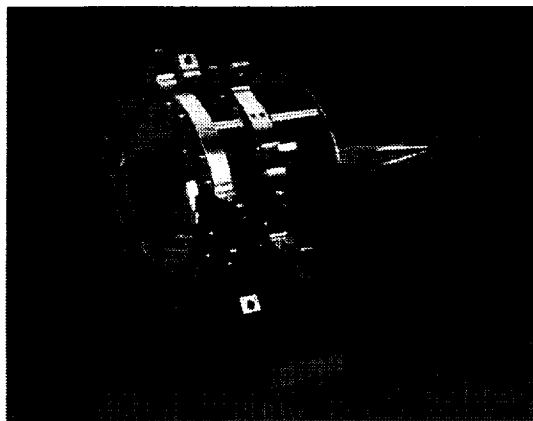


FIGURE 2. Refractive Secondary Concentrator

Stirling convertors have the potential to provide very high thermal-to-electric conversion efficiency. Stirling Technology Company (STC) in Kennewick, Washington has successfully designed, built, and operated free-piston convertors at 10 watts and 350 watts for terrestrial applications. The 350 watt STC convertor is pictured in Figure 3. STC is also developing a space-rated, 55 watt unit for radioisotope applications designed to provide system conversion efficiencies of greater than 24% (White, 1999). All of these engines share common technology characteristics including flexure bearings and linear alternators. The 10 watt engine has undergone endurance testing to over 50,000 hours in order to demonstrate long life and reliability.

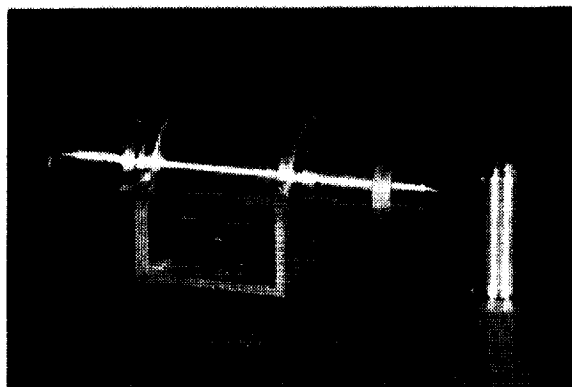


FIGURE 3. Free-Piston Stirling Convertor (courtesy of STC)

STUDY GROUND RULES

The overall study objectives were three-fold: 1) determine the feasibility of a solar Stirling power system for deep space missions using various advanced component technologies, 2) determine the key parameters which most influence system performance, and 3) compare system performance to other deep space power system options. The analysis evaluated system mass and deployed area for solar thermal power systems out to Pluto (about 40 AU).

Some of the key study assumptions are provided in Table 1. A reference electrical power level of 200 watts was chosen as typical of future deep space missions. The insolation and effective sink temperature were varied with distance from the Sun. Several of the component metrics were derived from design work performed by Orbital Sciences Corporation (OSC) in support of a Stirling concept for ARPS (Schock, 1999). The solar heat receiver was envisioned as a simple structure which supports the secondary concentrator and provides a thermal interface to the Stirling heater head, similarly to the GPHS container for the radioisotope Stirling concept. The waste heat radiator and the power management and distribution (PMAD) system were also derived from the OSC Stirling concept. A 10% system mass margin was included to account for interface structure and other unknowns.

TABLE 1. Key General Assumptions

Element	Assumptions
System	200 W electric power, 10% mass margin
Primary Concentrator	3.5% pointing loss, 10% wrinkle loss, 10% area margin
Secondary Concentrator	7:1 concentration ratio, reflective – 85% efficiency, refractive – 95% efficiency, passive cooling
Receiver	No thermal energy storage, 6 kg/kWt, 5% insulation loss
Stirling Convertor	STC design, 43 kg/kWe (includes active balancer), 2 convertors/system, temperature ratio <4.5, heater head temperature <1300 K, convertor efficiency = f(Trat)
Radiator	Heat pipe with C-C facesheets, 2 sided, 75% effective area, 6.4 kg/m ² , sink temperature = f(AU)
PMAD	28 Vdc bus, 150 W/kg, 95% efficiency

In order to determine the most promising component technologies, several different representative concentrator configurations were compared. These included a parabolic, thin-film inflatable system having a total reflection/transmission (R/T) efficiency of 63%, an areal density of 2 kg/m² (which includes the gas make-up system), and an Earth geometric concentration ratio (GCR) of 1600:1. The Earth GCR is defined as the ratio of the primary concentrator area to the receiver aperture area (or secondary entrance area) as required at 1 AU and provides a measure of the concentrator's overall surface accuracy. The theoretical maximum GCR for a solar concentrator at 1 AU having a focal distance-to-diameter (f/d) ratio of 1 is about 12000:1. This ratio varies with distance from the sun in relation to the subtended angle of the Sun relative to the concentrator. A second primary concentrator concept employing inflatable structure and a flat, fresnel reflector was assumed to have a combined R/T efficiency of 85%, an areal density of 0.5 kg/m², and an Earth GCR of 1000:1. Three different secondary concentrator options were considered: no secondary, a reflective secondary, and a refractive secondary. The masses of the secondary concentrators were scaled based on previous designs, the refractive crystal having a mass of over four times that of the reflective option for the same entrance diameter.

The Stirling temperature ratio (Trat, defined as T_{hot}/T_{cold}) and heater head temperature (T_{hot}) were concurrently optimized for minimum system mass. Generally, higher temperature ratios relate to higher conversion efficiency (smaller primary concentrators) at the expense of lower waste heat rejection temperatures (larger radiators). Consequently, a mass optimized temperature ratio results from the trade-off of concentrator mass and radiator mass. Varying the heater head temperature yields a minimum system mass based on a balance of infrared cavity loss and Stirling efficiency. Higher temperatures result in greater receiver losses, but allow the Stirling to operate at higher efficiency (higher Trat) without adversely effecting radiator size. Like the temperature ratio optimization, the heater head temperature optimization also results from a compromise between concentrator mass and radiator mass. For this study, Stirling temperature ratio was limited to 4.5 and heater head temperature was limited to 1300 K.

STUDY RESULTS

Figure 4 illustrates the Stirling optimization process showing system mass as a function of Stirling temperature ratio for a Jupiter mission (5.2 AU) using a Fresnel primary and a refractive secondary. Local minimum mass points for each of three different heater head temperatures are indicated by asterisks. Higher temperatures result in greater optimum temperature ratios. The global minimum mass design point was achieved at a heater head temperature of 1150K and a temperature ratio of 3.8 resulting in a system mass of 33.6 kg. The optimum heater head temperature and temperature ratio varied greatly with mission destination (i.e. solar distance). For Mars, minimum system mass was achieved at a heater head temperature of 1300K, while a Pluto system resulted in a minimum mass heater head temperature of 600K. The key factor in determining the optimum heater head temperature was the size of the receiver aperture and the associated infrared losses. At near-Earth distances, the receiver aperture was relatively small so a high temperature cavity did not produce excessive losses. However, as the primary concentrator increased for greater solar distances, a corresponding increase in receiver aperture size necessitated a lower cavity temperature to control receiver losses.

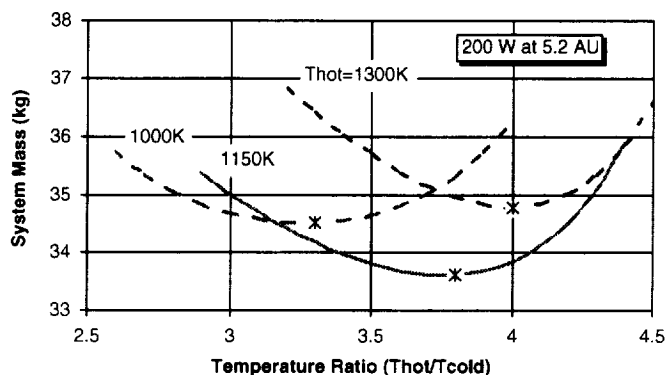


FIGURE 4. Stirling Mass Optimization for Jupiter Mission

A comparison of concentrator configurations for the 5.2 AU Jupiter mission is provided in Table 2. The Fresnel primary and refractive secondary combination resulted in the lowest system mass. The higher mass for the thin-film cases, was primarily a result of the 4x increase in areal density relative to the Fresnel. The Fresnel/refractive system also corresponded to the highest system efficiency, defined as the ratio of electric power produced to solar power collected by the primary. System efficiency was found to be a good indicator of system mass since the primary concentrator tended to be the dominant mass component. The primary concentrator was about 35% of the system mass at 5.2 AU, and beyond 10 AU, the mass fraction increased to greater than 50%.

TABLE 2. Concentrator Comparison at 5.2 AU

Primary	Secondary	Mass (kg)	Diam (m)	Thot (K)	Sys Eff (%)
Fresnel	None	40.9	6.1	750	13.5
	Reflective	34.4	5.6	1150	15.9
	Refractive	33.6	5.3	1150	18.2
Thin-film	None	90.2	6.0	900	13.9
	Reflective	83.4	6.0	1150	13.9
	Refractive	76.5	5.7	1150	15.7

The performance metrics assumed for the two primary concentrator options were chosen by projecting present day performance toward future systems. Uncertainty in those projections makes it appropriate to evaluate performance sensitivities. Figure 5 compares system mass versus primary diameter at 5.2 AU with parametric variations in Earth GCR, areal density, and R/T efficiency. The reference point represents the baseline assumptions for the Fresnel primary: 1000:1 Earth GCR, 0.5 kg/m², and 85% R/T efficiency. In general, the Earth GCR and the R/T efficiency parameters have a greater influence on the primary diameter size. Conversely, the concentrator areal density has a dramatic effect on system mass. These same trends were consistent over the entire range of solar distances considered in the study.

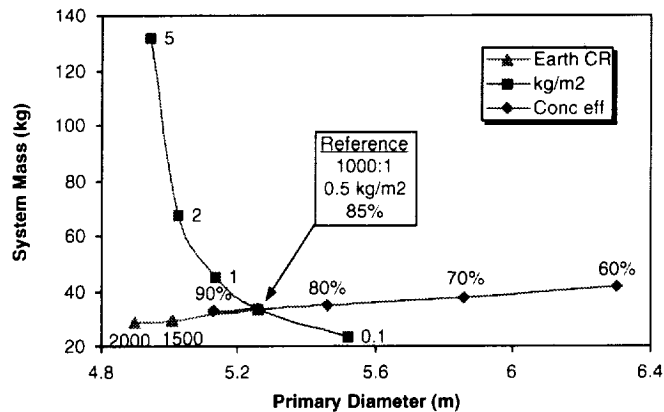


FIGURE 5. Primary Concentrator Sensitivity Analysis. 200 W at 5.2 AU

The variation in system mass and primary diameter with increasing solar distance is presented in Figure 6. Since areal density was determined to be a key system mass driver, values from 0.1 kg/m^2 to 5 kg/m^2 were considered. Based on the entrance diameter and the corresponding mass of the refractive crystal, it was desirable to use a reflective rather than a refractive secondary for missions beyond 10 AU. Below 10 AU, reasonable system mass was achievable with primary concentrators of less than 10 m and areal densities of less than 2 kg/m^2 . Systems for missions beyond 10 AU required primary concentrators greater than 20 m and areal densities below 0.5 kg/m^2 in order to achieve reasonable system mass.

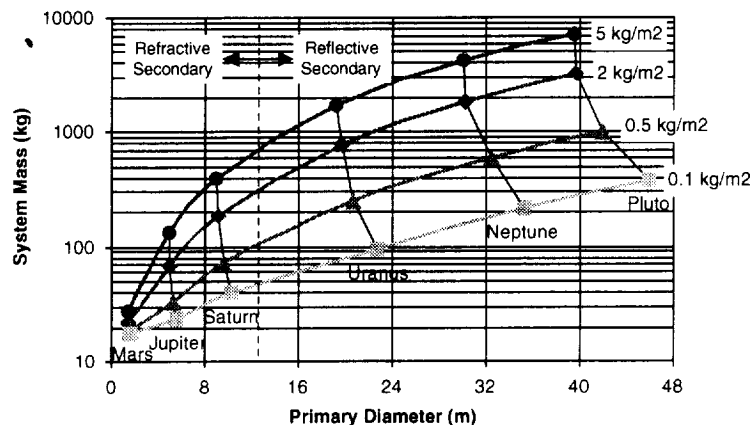


FIGURE 6. Performance Variations with Distance from Sun for 200 W Solar Stirling

SYSTEM COMPARISONS

Coincidentally, the 10 AU breakpoint serves as a reasonable upper limit for this technology as compared to radioisotope power systems. Table 3 compares system performance of 200 watt solar Stirling power systems at 1.5 AU (Mars), 5.2 AU (Jupiter), and 9.5 AU (Saturn) with two different radioisotope options: ARPS and small RTG. The solar power systems utilize the Fresnel/refractive concentrator configuration and vary in specific power from just under 3 W/kg for Saturn to almost 11 W/kg for Mars. The radioisotope systems require two units to approach the 200 watt end-of-mission (EOM) requirement resulting in specific power levels between 4 and 5 W/kg. A 200 watt solar Stirling for Jupiter has about the same mass as two ARPS units providing 150 watts EOM. The ARPS configuration would require 6 GPHS modules for the two units, while the small RTGs would require a total of 12 GPHS modules. In reference to the size of the solar collector, the 5.3 m primary concentrator diameter for the Jupiter system is similar to one Tracking and Data Relay Satellite System (TDRSS) antenna.

The natural decay of the radioisotope source causes a decrease in electrical power output with time. The EOM power and efficiency estimates for the radioisotope systems in Table 3 are based on the GPHS providing 232 watts per module after six years of operation (BOM thermal power from a GPHS module is 243 watts). Definition of the BOM power level for the solar systems requires further study. The large primary needed to collect power at the outer planets causes excessive power to be collected at Earth orbit. Some form of energy management would be required. Options include: 1) off-pointing of the primary, 2) adaptive focusing of the primary, 3) variable diameter shutter on the receiver aperture, or 4) a high temperature radiator for the receiver. An additional option might be to use a small solar panel for initial power and deploy the concentrator at further distances from the Sun.

TABLE 3. Comparison to Radioisotope Power Systems

	Solar Stirling	Solar Stirling	Solar Stirling	ARPS AMTEC	Small RTG
Distance (AU)	1.5	5.2	9.5	variable -	6 yr life
EOM power (W)	200	200	200	150	182
EOM system efficiency (%)	17.8	18.2	18.1	10.8	6.5
Total mass (kg)	18.6	33.6	74.4	29.8	45.2
Sp power (W/kg)	10.7	5.9	2.7	5.0	4.0
Prim dia (m)	1.6	5.3	9.7	-	-

CONCLUSION

The results of this study indicate that a solar Stirling power system is a feasible alternative for deep space applications. The key technology elements include a lightweight primary concentrator, a high efficiency secondary concentrator, and a high efficiency Stirling convertor.

System mass and deployed area were characterized out to 40 AU. Various concentrator configurations were considered included Fresnel and thin-film primaries, and reflective and refractive secondaries. Earth geometric concentration ratio and reflection/transmission efficiency of the primary were found to have a major effect on concentrator deployed area. Concentrator areal density was a key system driver, having a dominant influence on overall system mass. The flexibility of the Stirling convertor to operate at variable heater head temperatures, depending on the mission destination, helped to control infrared losses and maintain high overall system efficiency.

The solar Stirling system compared favorably with other deep space power options. Mass was competitive with radioisotope power systems out to 10 AU. A 200 watt solar power system for Jupiter offered 18.2% system efficiency at a specific power of 5.9 W/kg, a modest improvement over projected ARPS performance without the complications brought on by nuclear sources.

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REPORT DOCUMENTATION PAGE			Form Approved OMB No. 0704-0188	
Public reporting burden for this collection of information is estimated to average 1 hour per response, including the time for reviewing instructions, searching existing data sources, gathering and maintaining the data needed, and completing and reviewing the collection of information. Send comments regarding this burden estimate or any other aspect of this collection of information, including suggestions for reducing this burden, to Washington Headquarters Services, Directorate for Information Operations and Reports, 1215 Jefferson Davis Highway, Suite 1204, Arlington, VA 22202-4302, and to the Office of Management and Budget, Paperwork Reduction Project (0704-0188), Washington, DC 20503.				
1. AGENCY USE ONLY (Leave blank)		2. REPORT DATE December 1999		3. REPORT TYPE AND DATES COVERED Technical Memorandum
4. TITLE AND SUBTITLE Solar Stirling for Deep Space Applications			5. FUNDING NUMBERS WU-632-1A-1K-00	
6. AUTHOR(S) Lee S. Mason				
7. PERFORMING ORGANIZATION NAME(S) AND ADDRESS(ES) National Aeronautics and Space Administration John H. Glenn Research Center at Lewis Field Cleveland, Ohio 44135-3191			8. PERFORMING ORGANIZATION REPORT NUMBER E-12012	
9. SPONSORING/MONITORING AGENCY NAME(S) AND ADDRESS(ES) National Aeronautics and Space Administration Washington, DC 20546-0001			10. SPONSORING/MONITORING AGENCY REPORT NUMBER NASA TM-1999-209656	
11. SUPPLEMENTARY NOTES Prepared for the Space Technology and Applications International Forum-2000 sponsored by the American Institute of Physics, Albuquerque, New Mexico, January 30-February 3, 2000. Responsible person, Lee S. Mason, organization code 5490, (216) 977-7106.				
12a. DISTRIBUTION/AVAILABILITY STATEMENT Unclassified - Unlimited Subject Category: 20 This publication is available from the NASA Center for AeroSpace Information, (301) 621-0390.			12b. DISTRIBUTION CODE	
13. ABSTRACT (Maximum 200 words) A study was performed to quantify the performance of solar thermal power systems for deep space planetary missions. The study incorporated projected advances in solar concentrator and energy conversion technologies. These technologies included inflatable structures, lightweight primary concentrators, high efficiency secondary concentrators, and high efficiency Stirling convertors. Analyses were performed to determine the mass and deployed area of multi-hundred watt solar thermal power systems for missions out to 40 astronomical units. Emphasis was given to system optimization, parametric sensitivity analyses, and concentrator configuration comparisons. The results indicated that solar thermal power systems are a competitive alternative to radioisotope systems out to 10 astronomical units without the cost or safety implications associated with nuclear sources.				
14. SUBJECT TERMS Solar energy conversion; Stirling engine; Solar collectors			15. NUMBER OF PAGES 12	
			16. PRICE CODE A03	
17. SECURITY CLASSIFICATION OF REPORT Unclassified	18. SECURITY CLASSIFICATION OF THIS PAGE Unclassified	19. SECURITY CLASSIFICATION OF ABSTRACT Unclassified	20. LIMITATION OF ABSTRACT	