



Flow Analysis of X-34 Main Propulsion System Feedlines

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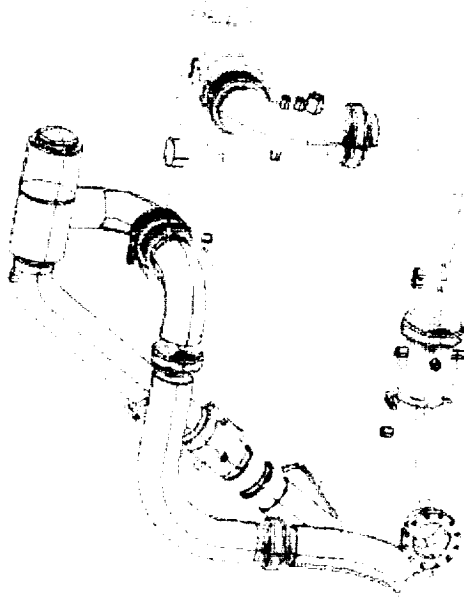
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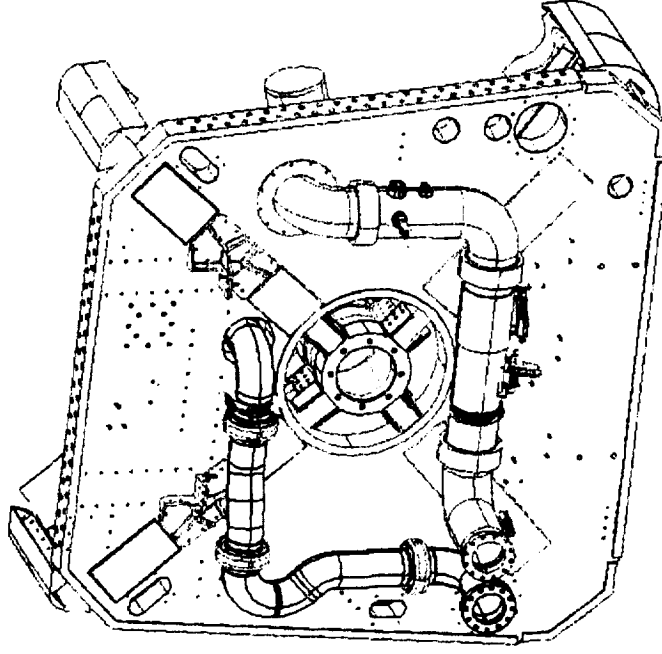
Flow Analysis of X-34 MPS Feedlines



- Objective -
 - 1) Determine the flow development at the engine interface
 - 2) Predict the pressure drop in the feedline.



Old Configuration (MSFC's Design, Phase 2, 10/98)

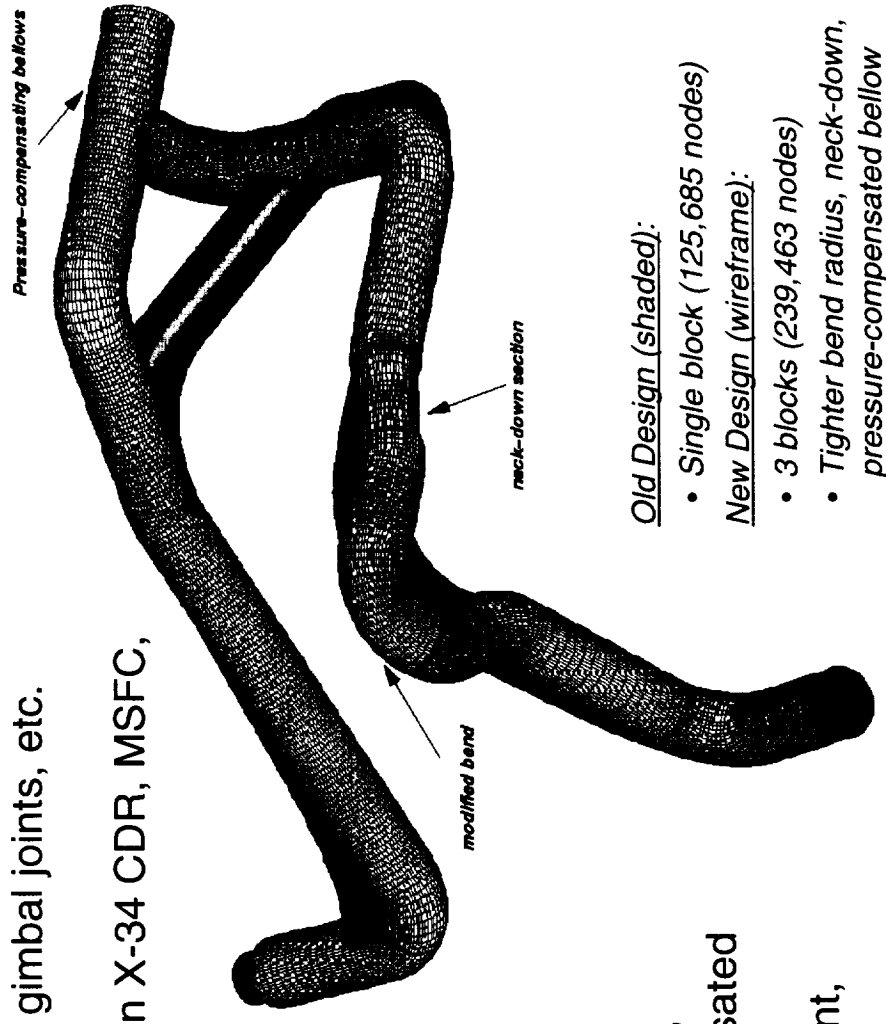


New Configuration (OSC's Design, 10/99)



Computational Grids

- Assumptions -
 - 1) Disregard flow meters, fuel filters, gimbal joints, etc.
 - 2) Fully-developed flow at the inlet
 - 3) Reference conditions are based on X-34 CDR, MSFC, 12/15-17/97



- Method -

- 1) Generate grid based on the IGES geometry provided by OSC, 10/98
- 2) Model includes pressure-compensated bellows
- 3) Perform single phase, 3D, turbulent, incompressible analysis

Old Design (shaded):

- Single block (125,685 nodes)

New Design (wireframe):

- 3 blocks (239,463 nodes)
- Tighter bend radius, neck-down, pressure-compensated bellows

- Solve the curvilinear coordinate transport equations:

$$(1/J)(\partial \rho q / \partial t) = \partial [-\rho U_i q + \mu G_{ij} (\partial q / \partial \xi_j)] / \partial \xi_i + (1/J) S_q$$

where

$$J = \partial(\xi_i \xi_j) / \partial(x, y)$$

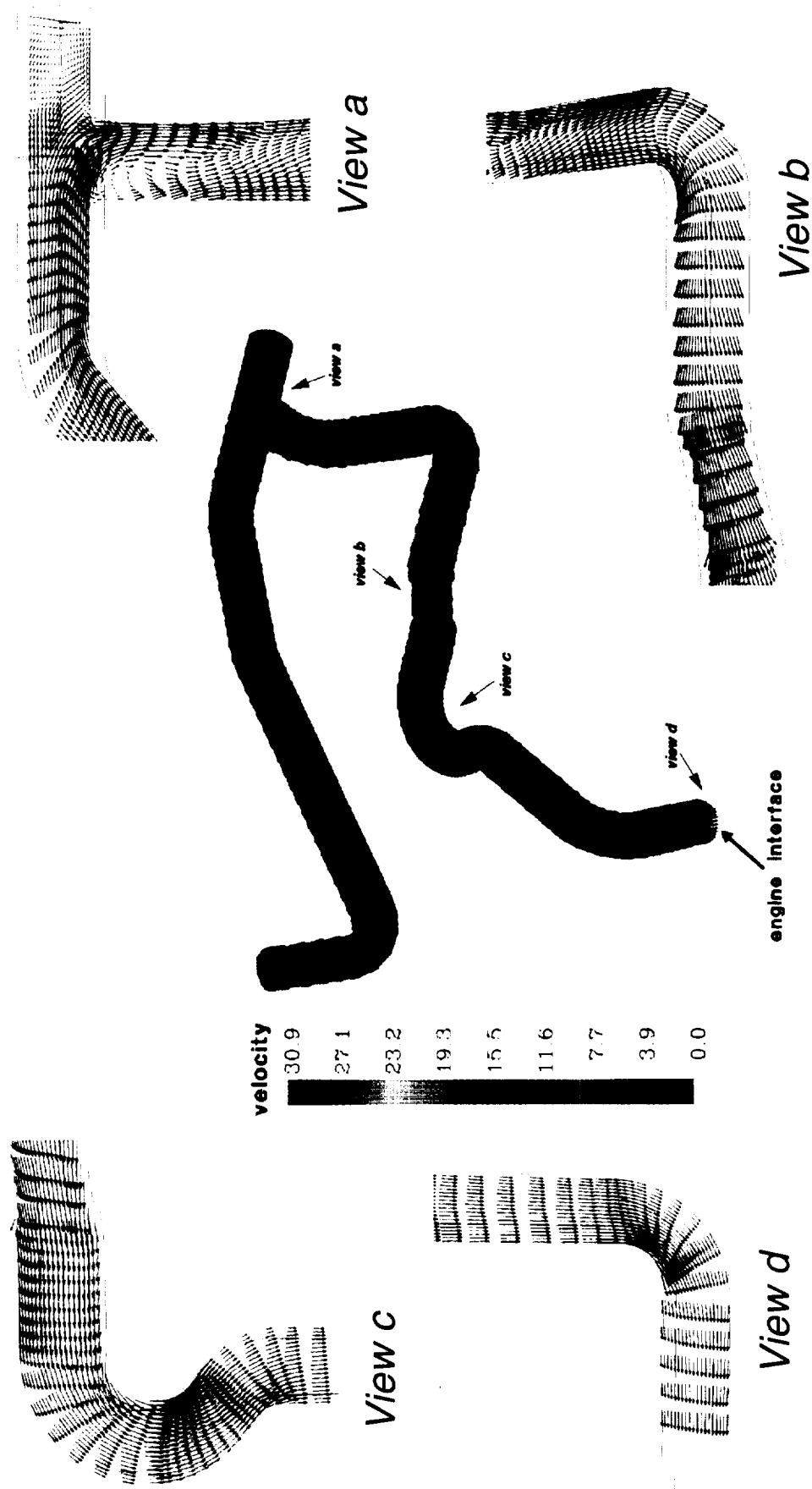
$$U_i = (u_i / J) (\partial \xi_i / \partial x_j)$$

$$G_{ij} = (\partial \xi_i / \partial x_k) (\partial \xi_j / \partial x_k) / J$$

- Finite-difference approximations are used to establish a system of linearized algebraic equations
- Relaxation schemes are based on 2nd and 4th order central differencing with artificial dissipation



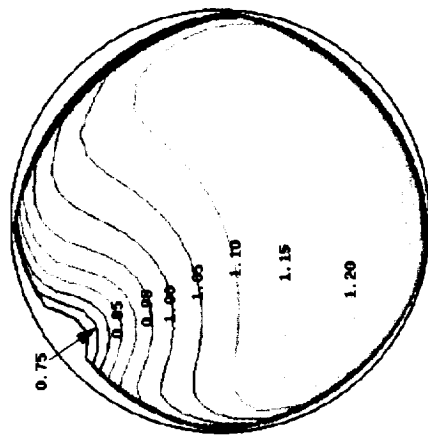
Velocity Vectors



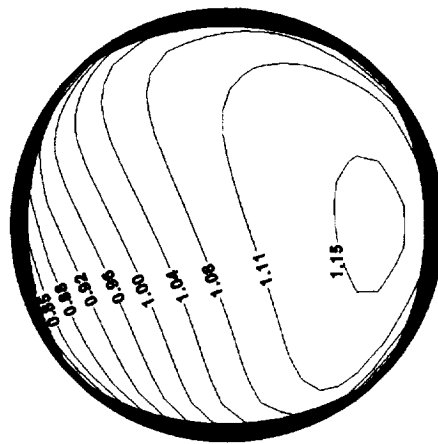
Velocity at the Engine Interface



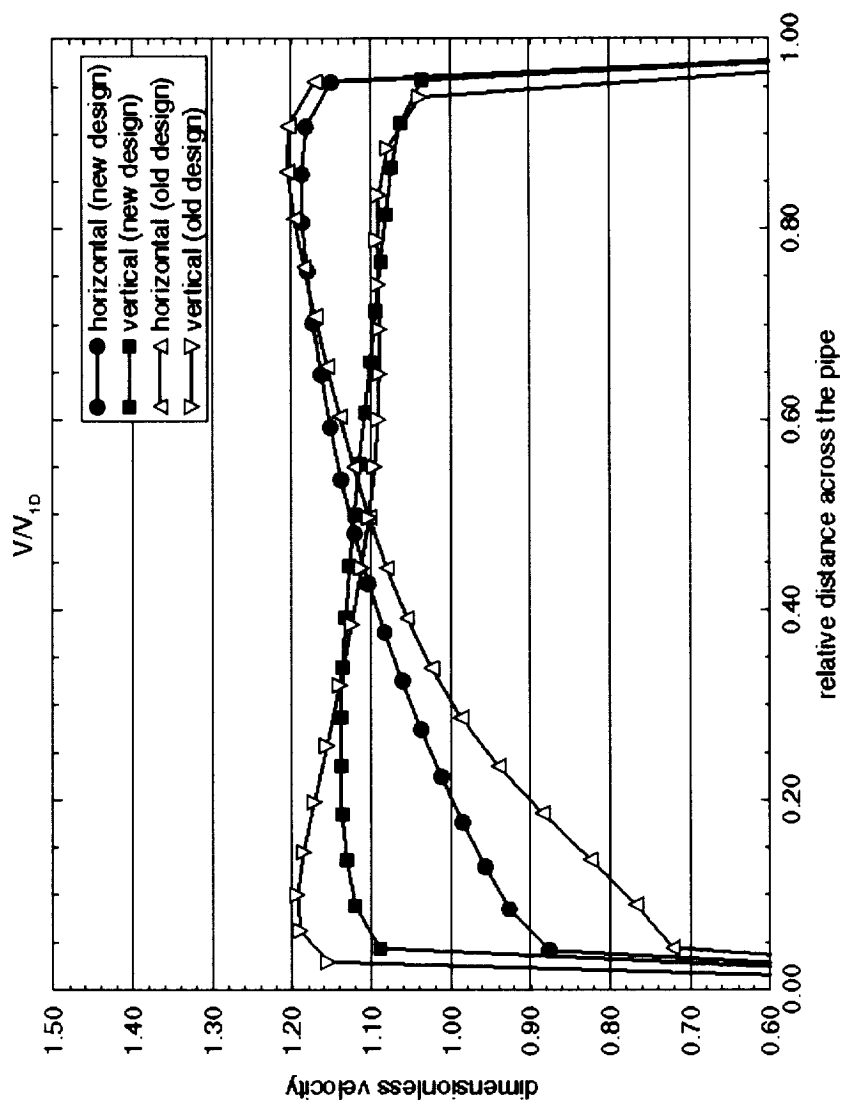
RP-1 Feedline CFD Results (MSFC's Design)
Nondimensionalized Axial Velocity



RP-1 Feedline CFD Results: Orbital's Design
Nondimensionalized Axial Velocity at the Engine Interface



X-34 RP-1 Feedline CFD Results

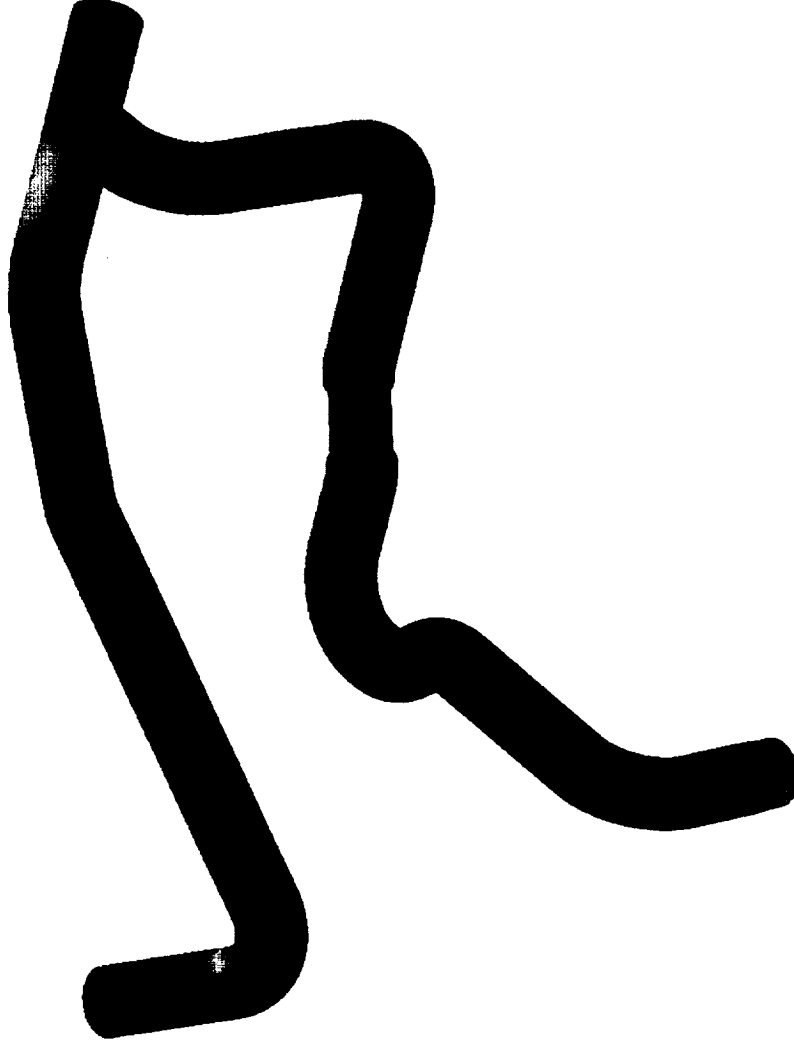


Static Pressure



pressure

42.6
41.5
40.4
39.3
38.2
37.1
36.0
35.0
33.9

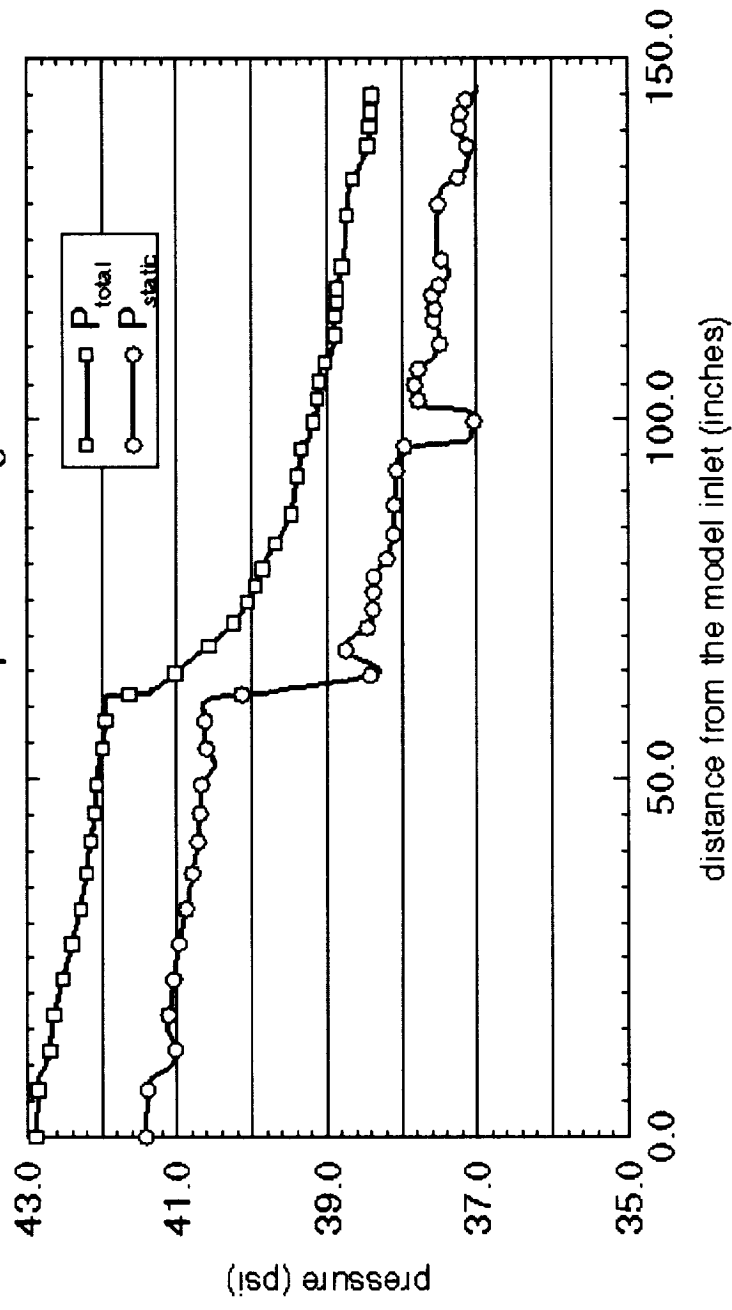


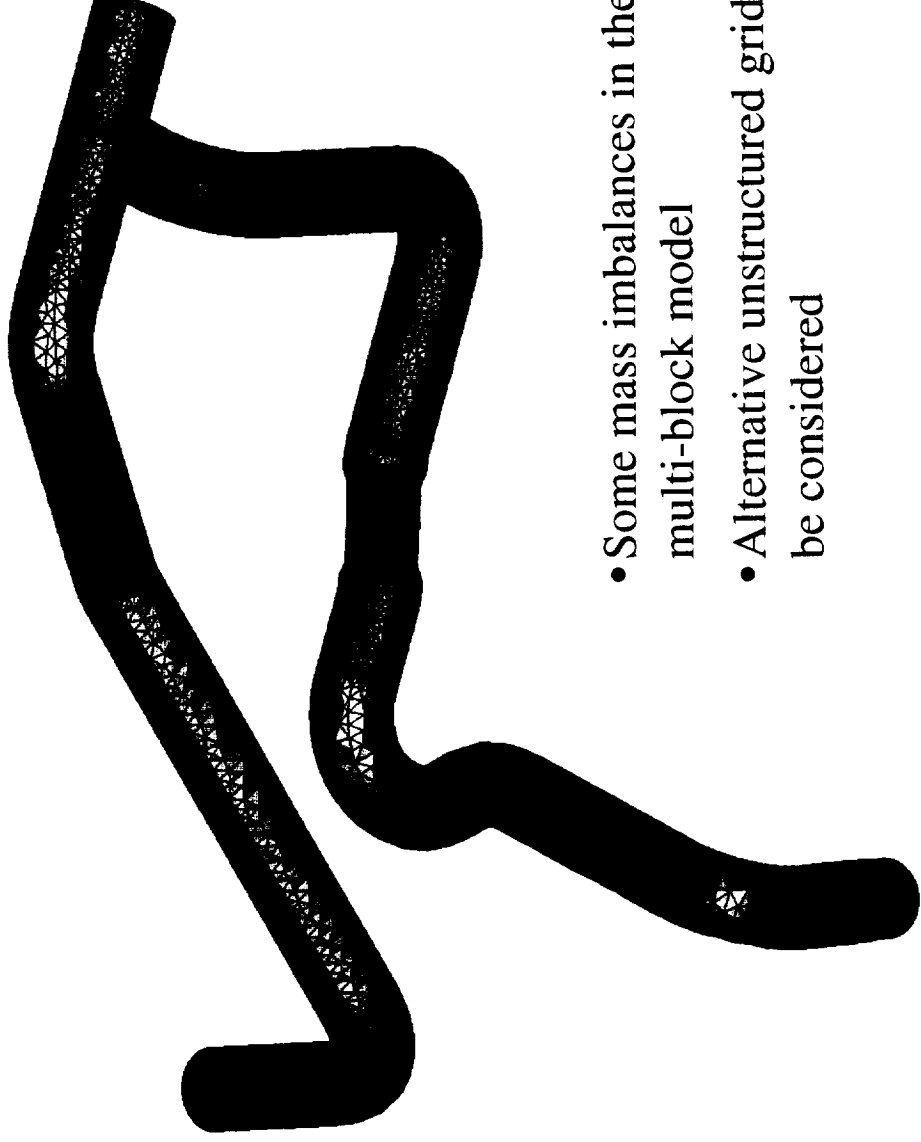
engine interface



RP 1 Feedline Mass Averaged Pressure

Referenced to 37 psi. at the engine interface





- Some mass imbalances in the structured, multi-block model
- Alternative unstructured grid model may be considered

Results and Conclusion



Results -

- 1) Flow separation can be seen in the stagnation region only
- 2) No flow distortions at the engine interface
- 3) Pressure drop ($\Delta P = 4.6$ psi) is within acceptable range

Conclusion -

- 1) New duct design does not have significant affect on the outlet flow
- 2) No flow straighteners are required
- 3) Flow development is more favorable in new model although pressure drop is slightly higher

