



Subscale Test Methods for Combustion Devices

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Outline

- Motivation for Scaled Experiments
- Brief Scaling History
 - Steady-State Combustion
 - Combustion Stability
 - Life Prediction
- Scaling Approaches Presently Used at Purdue

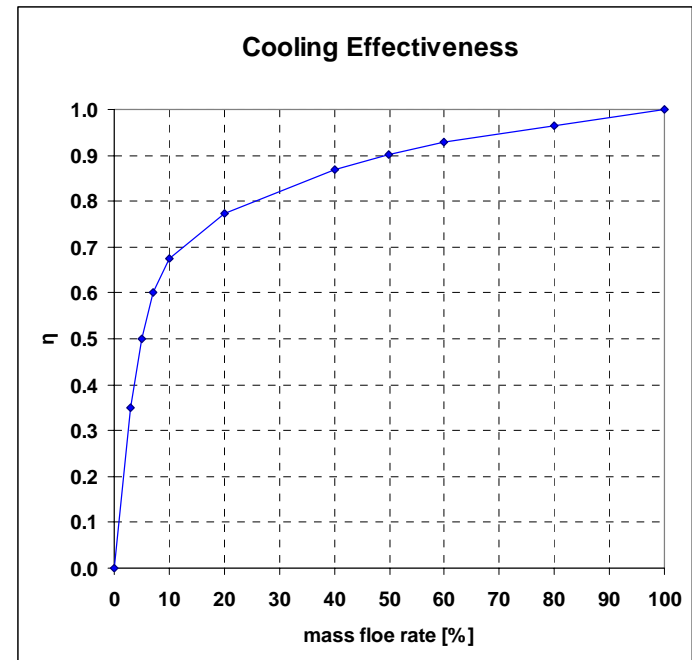
Background

- Stated goals for long-life LRE's have been between 100 and 500 cycles
 - Inherent technical difficulty of accurately defining the transient and steady state thermochemical environments and structural response (strain)
 - Limited statistical basis on failure mechanisms and effects of design and operational variability
 - Very high test costs and budget-driven need to protect test hardware (aversion to test-to-failure)
- Ambitious goals will require development of new databases
 - Advanced materials, e.g., tailored composites with virtually unlimited property variations
 - Innovative functional designs to exploit full capabilities of advanced materials
 - Different cycles/operations
- Subscale testing is one way to address technical and budget challenges
 - Prototype subscale combustors exposed to controlled simulated conditions
 - Complementary to conventional laboratory specimen database development
 - Instrumented with sensors to measure thermostructural response
 - Coupled with analysis

SSME Film Cooling Analysis

- Configuration
 - Propellant = LOX + LH2 with O/F = 6.02
 - $M_{\dot{LOX}} = 64,000$ liter/min
 - $M_{\dot{LH2}} = 178,000$ liter/min
 - $M_{\dot{coolant}}$ for regen cooling = 29.06 lb/sec
- Chamber condition
 - $P_c = 3300$ psi
 - $T_c = 3500$ K (5840 F)
 - $D_{throat} = 10.88''$
 - $E = 77$
- Cooling channel
 - Wall thickness = 0.03''
 - Width = 0.04 ''
 - Height = 0.12 ''
 - $P_{throat} = 3851$ psi
- Thermal condition at throat
 - Heat flux = 80 Btu/in²-s
 - $hg = 58000$ W/m²-K
 - $T_{wg} = 1100$ F
- Wall adiabatic temperature
 - $T_{aw} = T_r - \eta(T_r - T_{co})$
 - Where T_r = recovery temperature
 - η = film cooling efficiency
 - T_{co} = initial coolant temperature

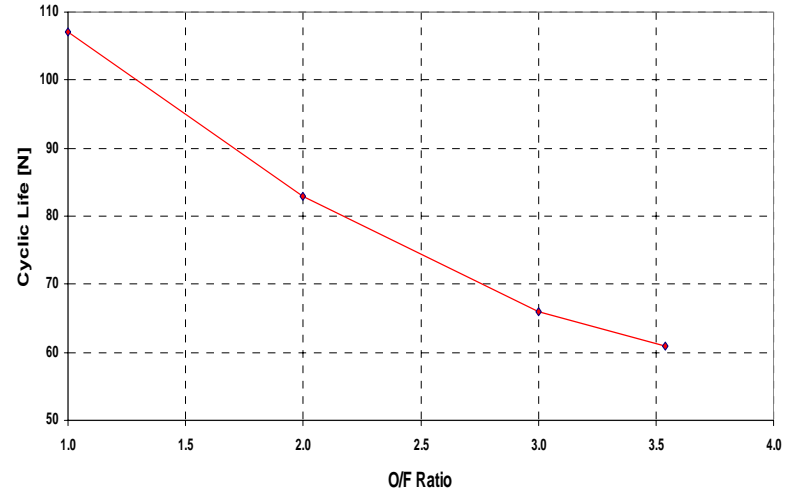
- Current near wall O/F ratio
 - $\dot{q} = hg(T_{aw} - T_{wg})$
 - Where $\dot{q} = 80$ Btu/in²-s
 - $hg = 58000$ W/m²-K
 - $T_{wg} = 1100$ F
 - $T_{aw} = 3125$ K
 - $\eta = 0.5$
 - $T_{co} = 2750$ K
 - O/F_{nw} = 3.54 from Flame temperature vs O/F ratio chart



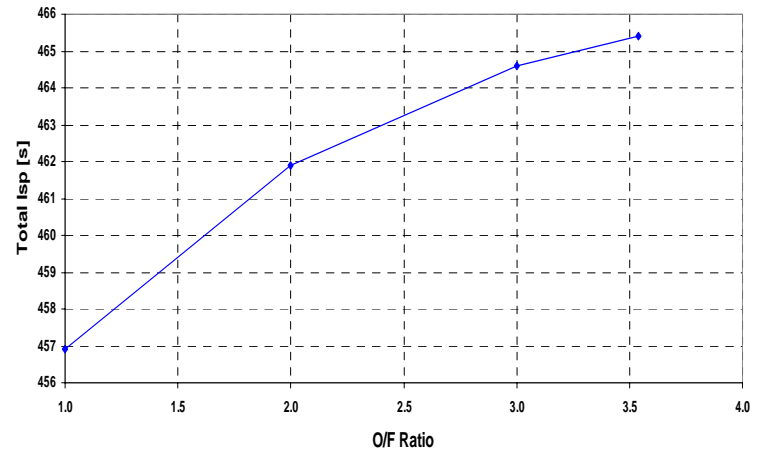
SSME Film Cooling Analysis

- Current film cooling condition
 - $O/F_{nw} = 3.54$
- Parametric study with fixed film flow rate (5 %)
 - *Porowski et al. method (AIAA Journal Vol. 2 No. 2, 1985)
 - O/F_{nw} change = $3.54 \rightarrow 1.0$
 - Life change = $61 \rightarrow 107$ (75.4% increase)
 - Isp change = $465 \rightarrow 457$ (1.83 % decrease)

SSME O/F vs Life



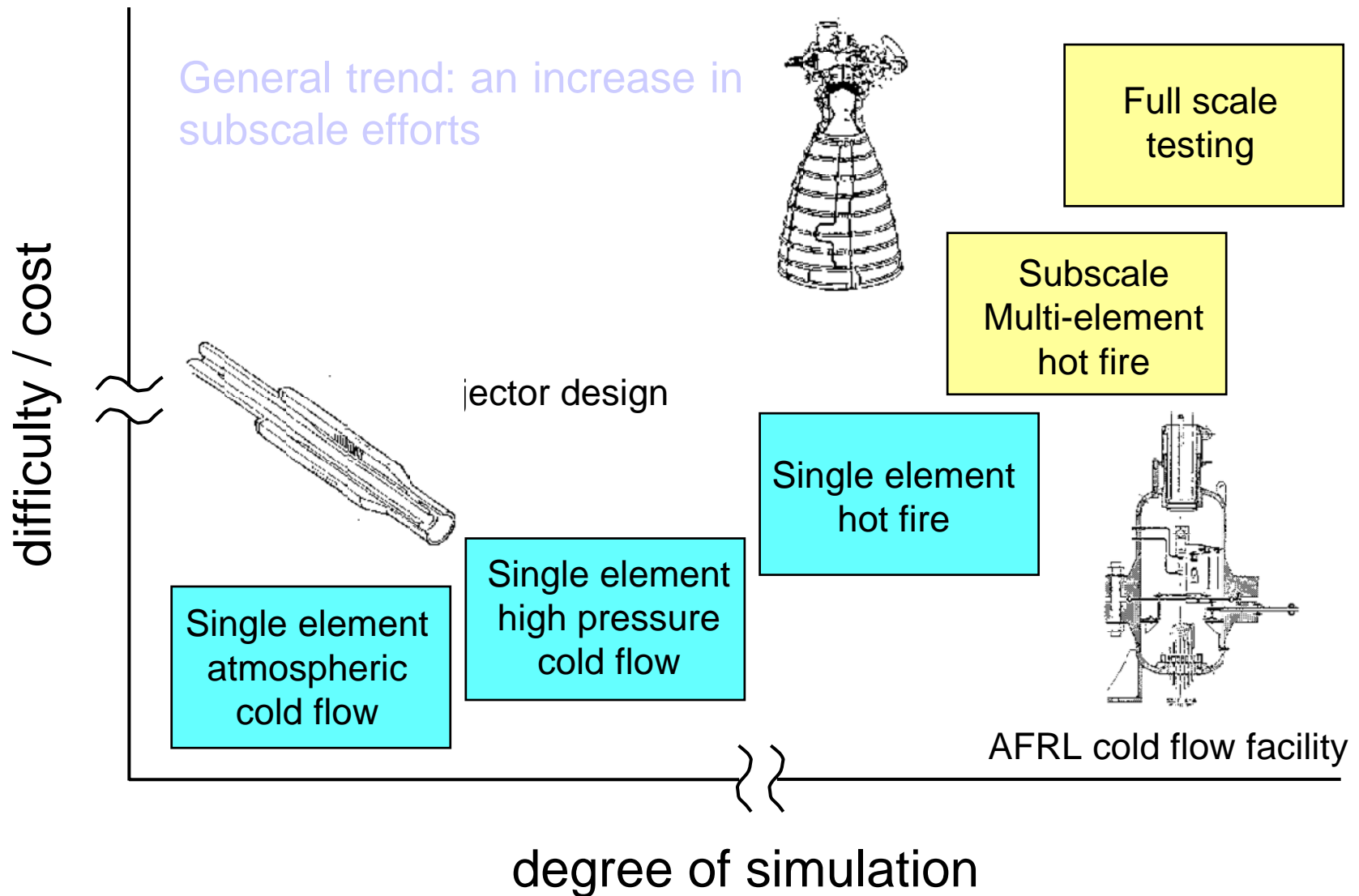
Isp vs O/F Variation
(coolant $\dot{m} = 5.0\%$)



Scaling Objectives and Approaches

- Combustor characterization is goal
 - Validation data for design analysis models
 - Assess innovative functional design, materials, operation
 - Investigations into specific physics
- Single element, multi-element, 40K, 250K
- Cold flow and hot fire
- Performance, heat transfer, life, stability
- Experimental objective needs to define scaling approach and measurement
 - Well-instrumented combustors linked to analysis
 - Thrust level and number of elements
 - Element scaling and configuration

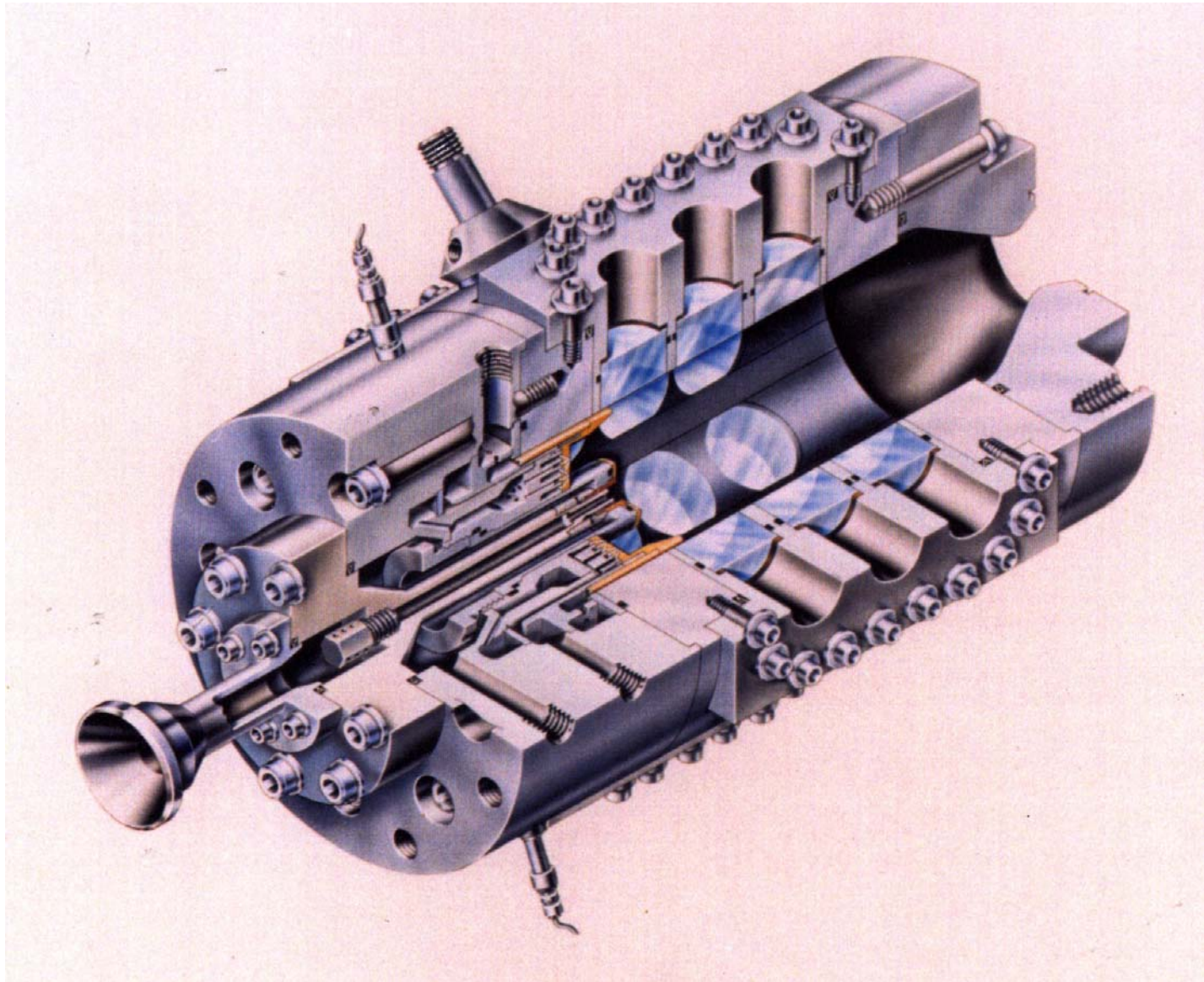
Hierarchy of injector experiments



Brief History of Scaling in the US – Steady State Combustion

- JPL studies of mixing efficiencies of impinging jets
- Bell Aerospace/AFRL holographic and shadowgraphic studies of combusting flows
- Rocketdyne development of LISP methodology for SDER
- Aerometrics development of PDPA
- Rocketdyne studies of flameholding behind LOX post
- PSU measurements of chemical species in HO combustors
- AFRL studies of supercritical jets

Single Element Test Chamber



Stability Scaling

- Simulation of chamber dynamics in subscale configuration is very difficult
 - Acoustic frequencies scale as $\sim 1/d$
 - Pressure ν velocity sensitivity
- Scaling approaches
 - Wedges, T-burners, 2-d chambers
 - $1T = 3T$ scaling
- Single element rarely used in US, but is more typical in Russia

Experimental Approach of Bazarov

This facility screened Injector elements for Liq/liq and gas/liq Injectors for over 20 Years (1965-85)

Typical $P_c = 750$ psi,
Total flowrate of 5 lb/s

‘self-oscillation’ and response to pulsations measured

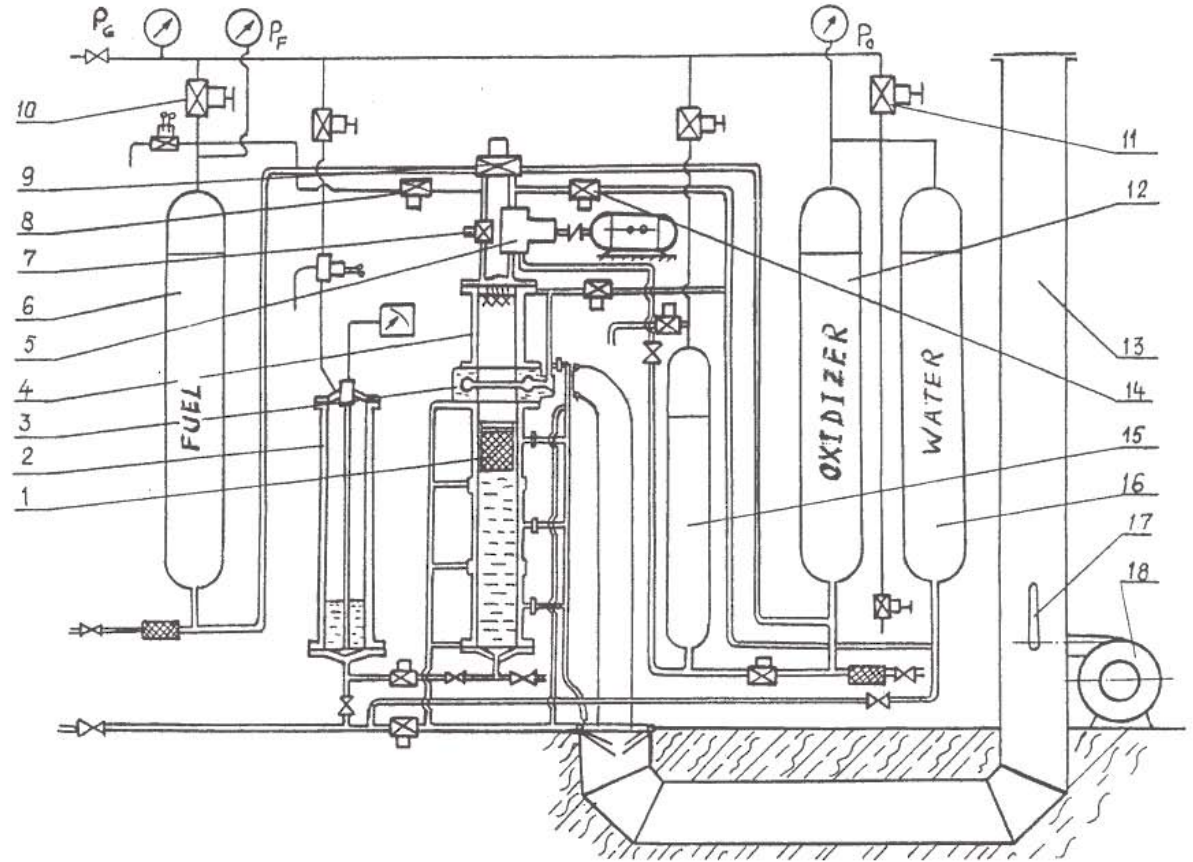


Fig.8 Pneumatic and hydraulic scheme of fire stand

1-piston, 2-measuring vessel, 3-nozzle collector, 4-combustion chamber, 5-pulsator, 6-fuel tank, 7-time delay valve, 8-blow through valve, 9-main bi-propellant valve. 10,11-pressurising gas reducers, 12-oxidizer tank, 13-exhaust tubes, 14-water valve, 15-oxidizer return tank, 16-pressurised water tank, 17-ejector, 18-air compressor

Experimental Approach of NIICHIMMASH

- Use full-scale injector elements
- Experiment designed to simulate controlling process
 - mixing
- Match equivalence ratio and volumetric flowrates using diluted gaseous propellants
- Combustor acoustics matched by using appropriately sized low-pressure chamber
- Stability boundaries determined by varying flowrates
- Relative boundaries indicate stability ranking

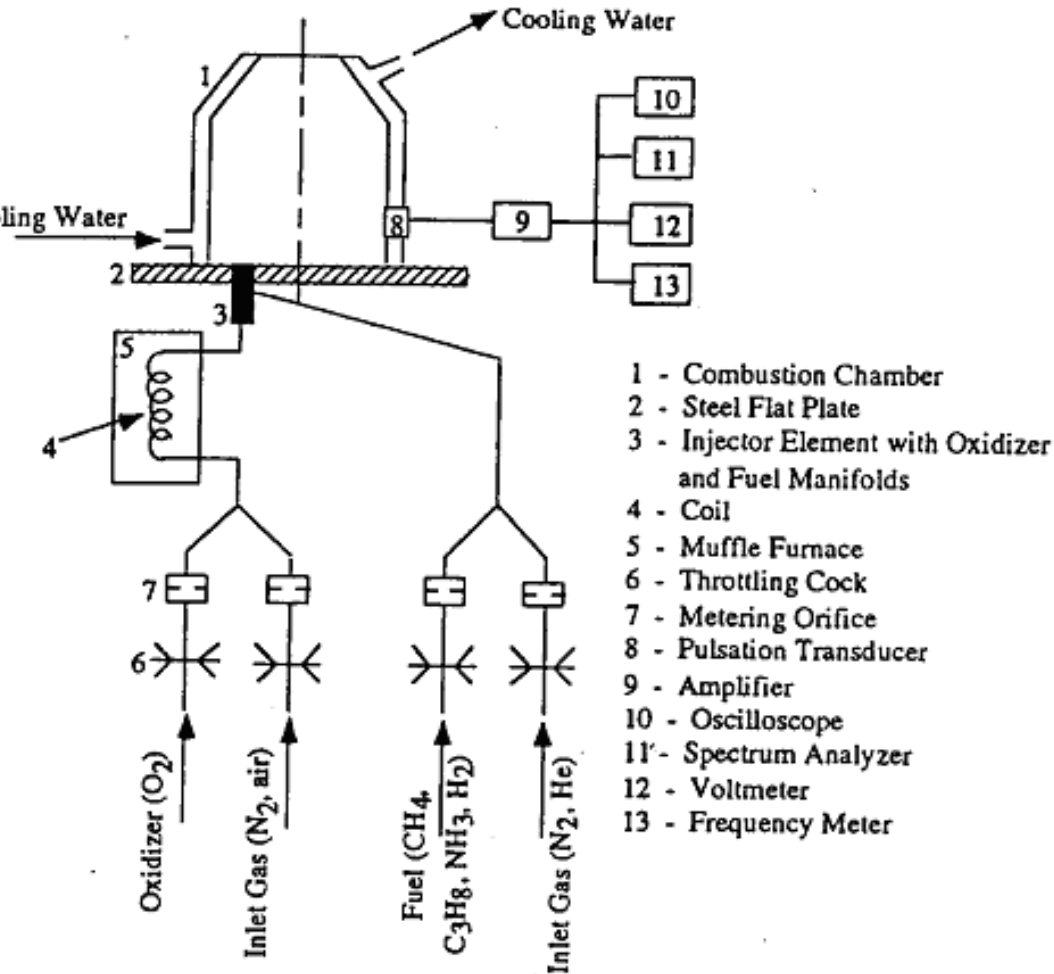


Figure 6. Schematic of Single Element Model Set-up and Instrumentation

Propellant Distribution Effects

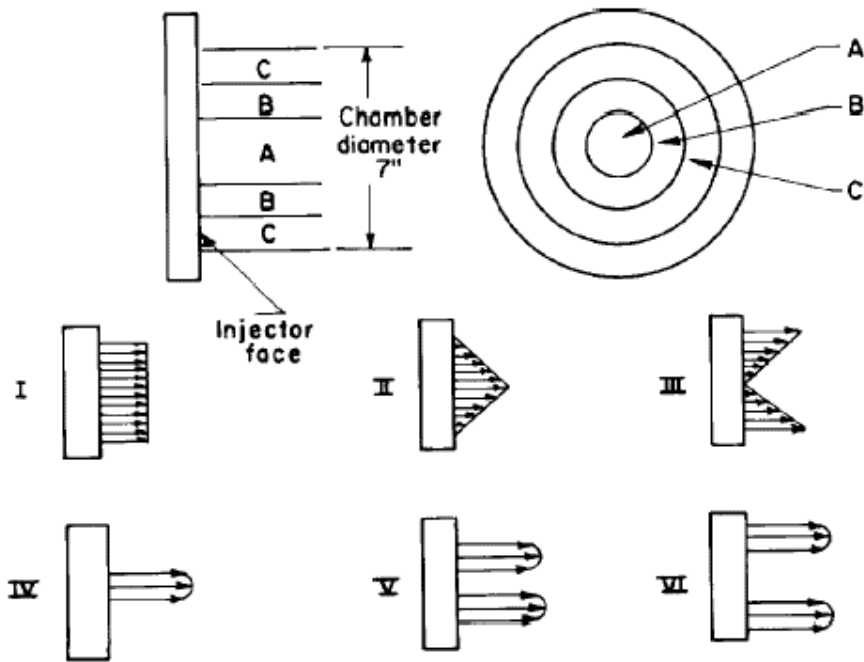


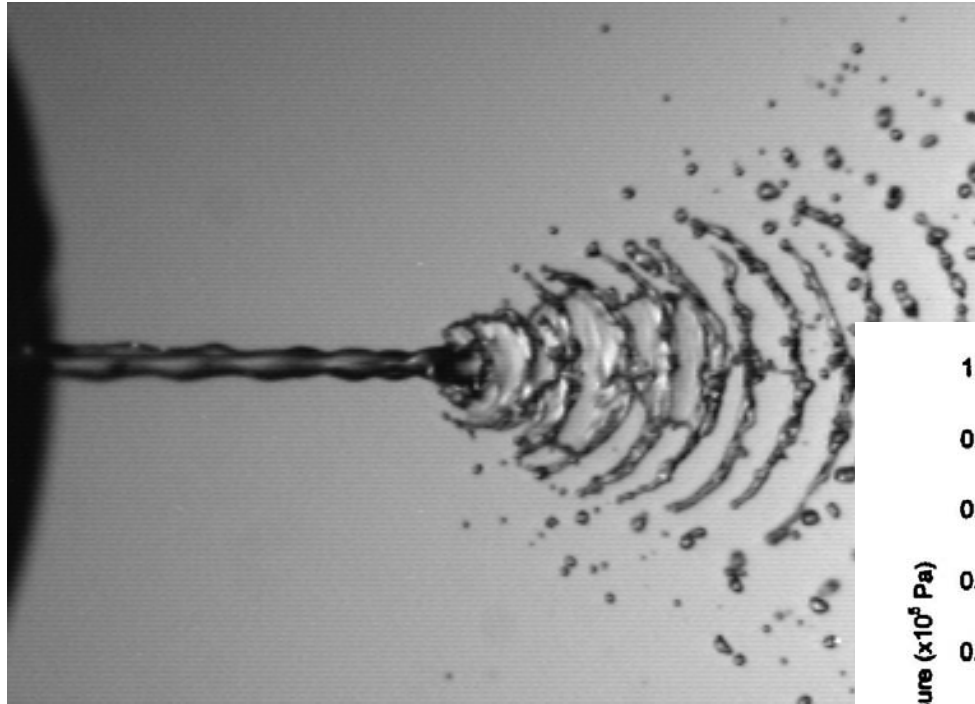
FIGURE 7.2.5a.—Injection radial profile comparison.

TABLE 7.2.5a.—GAS ROCKET TEST HISTORY WITH VARIOUS INJECTION PROFILES

[Instabilities initiated spontaneously and linearly; mean chamber pressure, 150 psia; combustion chamber diameter, 7 in.; combustion chamber length, 6 in.]

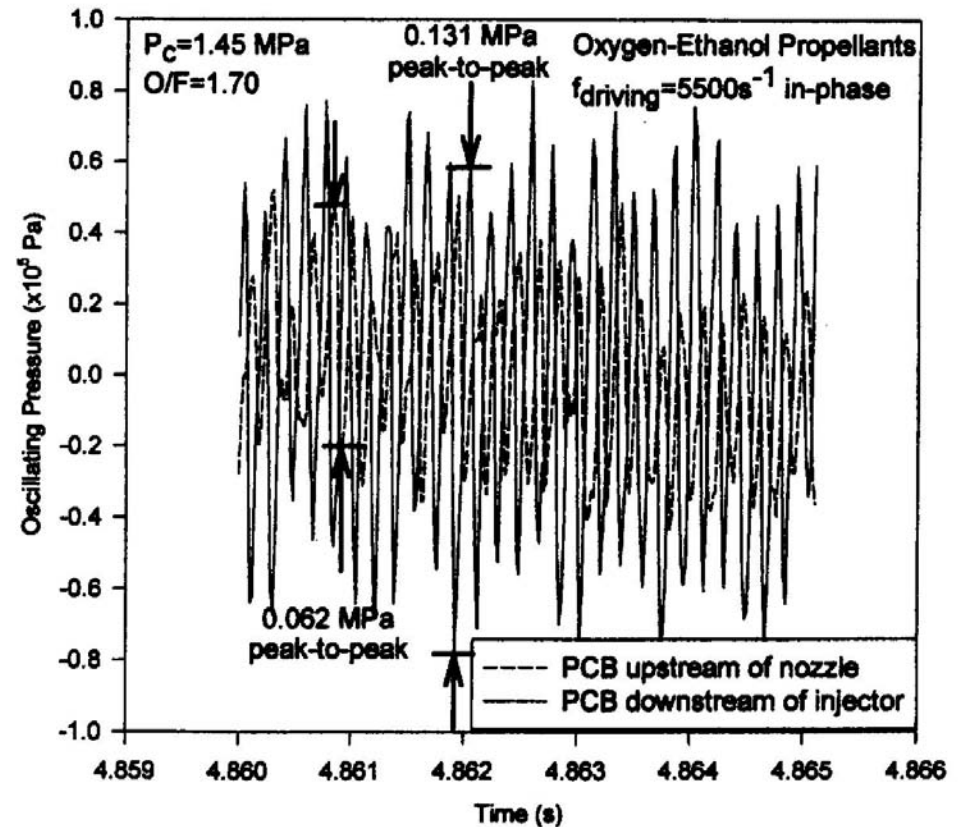
| Profile | Amplitude, psi | Mode |
|---------|----------------|----------------|
| I | 7 | 1st tangential |
| II | 0 | Stable |
| III | 11 | 1st tangential |
| IV | 13 | 1st radial |
| V | 0 | Stable |
| VI | 40 | 1st tangential |

Single Element 'Instability'

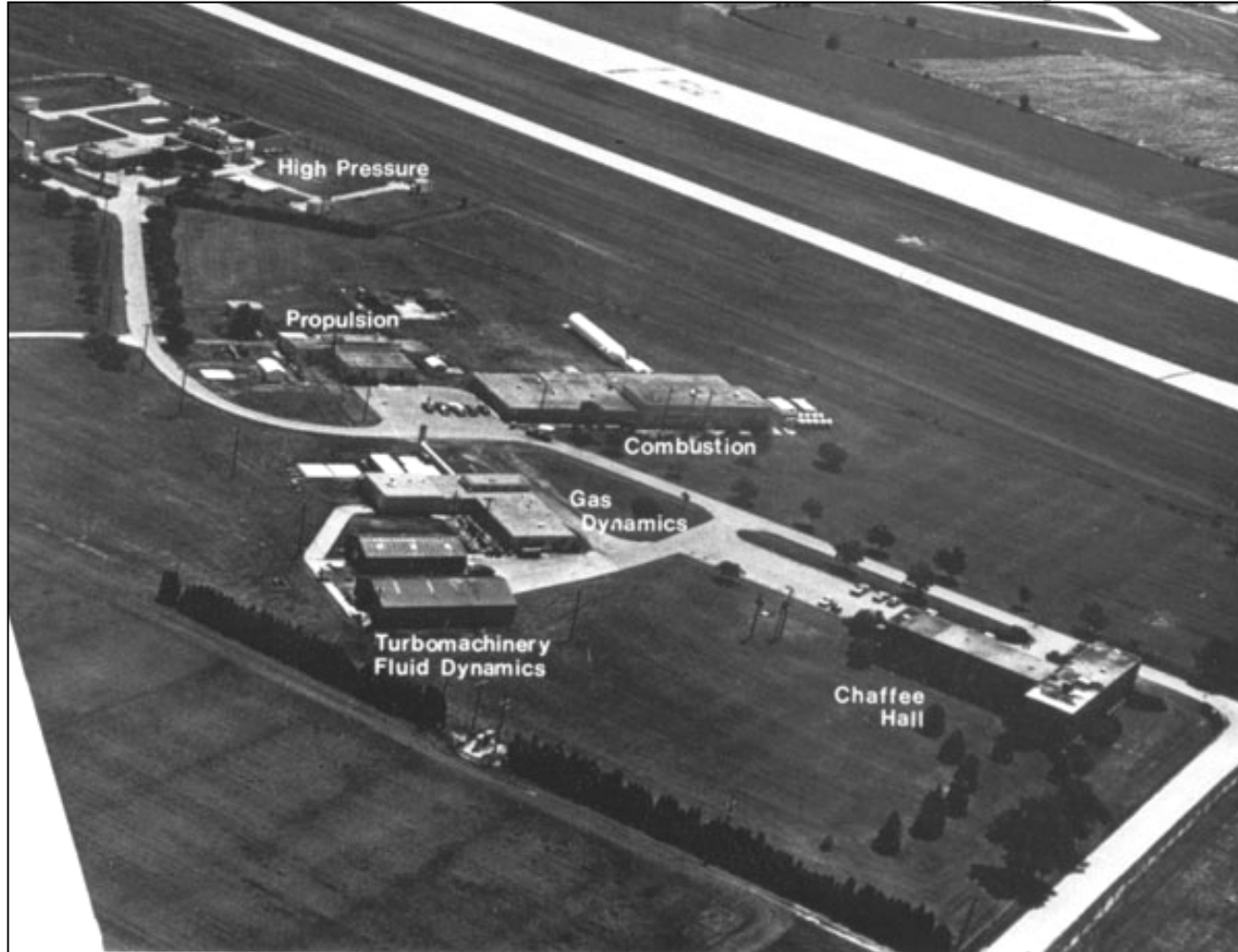


Combustor oscillations at driven atomization frequency

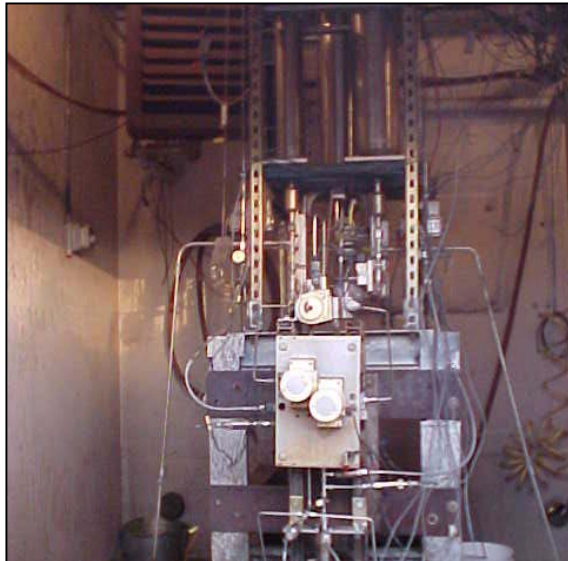
Impinging jets driven by piezoelectric actuator



Subscale Test Activities at Purdue - Maurice Zucrow Laboratory



Advanced Propellants and Combustion Lab



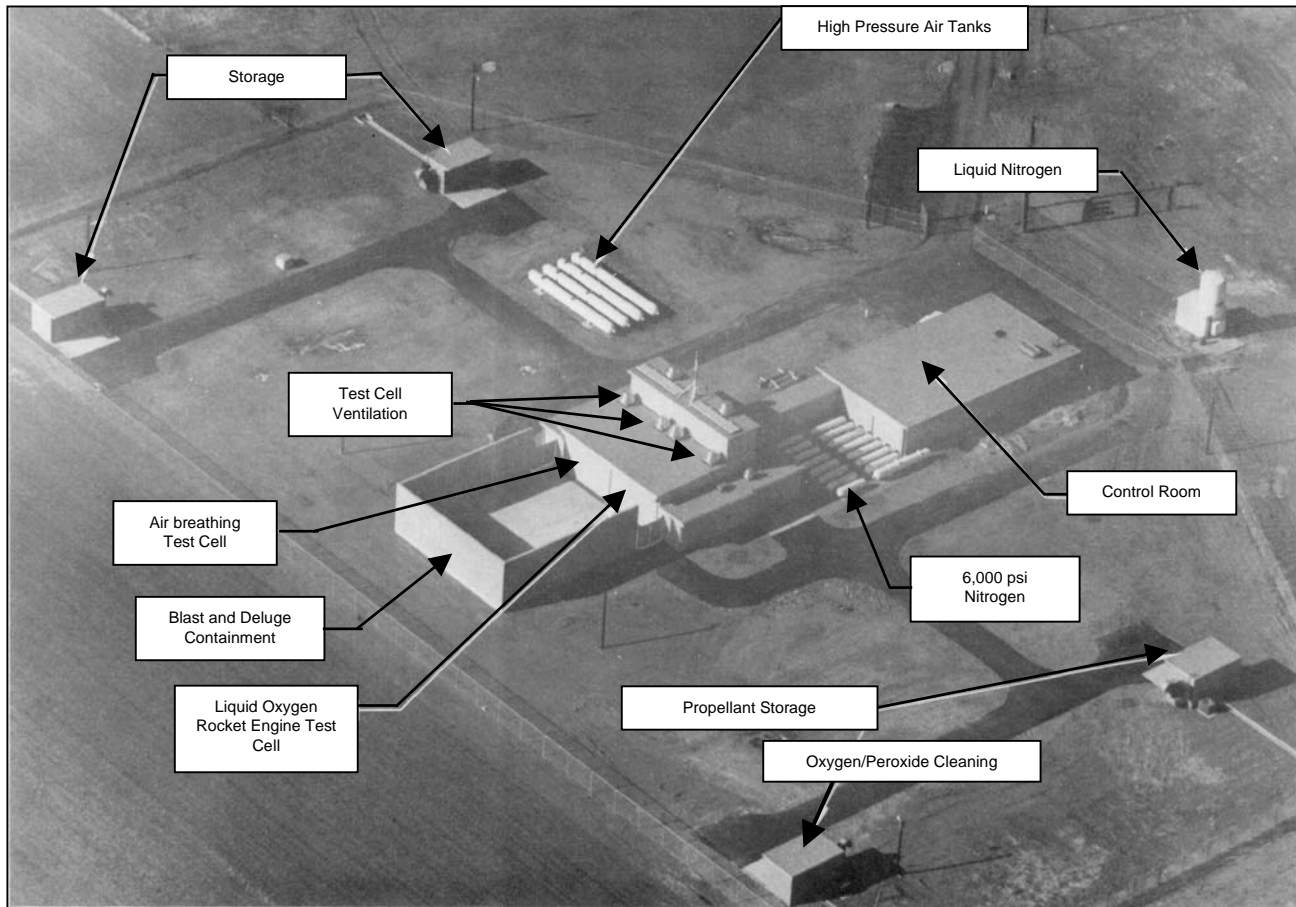
- Two cells w/ 1 Klbf thrust stands
- Propellant supply of 1800 psia
- 2 - 4 gallon oxidizer tanks
- 1 & 4 gallon fuel tanks
- National Instruments hardware & LabView software
 - 32 channels pressure
 - 32 channels temperature



- All valves computer controlled
- Rapid test article installation
- Design/Build/Test course

High Pressure Lab

Renovation funded thru Indiana 21st Century R & T Fund –
Propulsion and Power Center of Excellence
Facility activated in May '03



6,000 psi Nitrogen System



- Pressurization, Actuation and Purge Gas
- 2,400 gallon Liquid Nitrogen Tank w/ 6,000 psi Pump
- 253 ft³ 6,000 psi Nitrogen Tube Trailer
- Computer Controlled Pressurization Systems

Propellant/Coolant Tanks



- 22 gal 5,000 psi LOx
- 16 gal 5,000 psi Fuel
- 220 gal 5,000 psi H₂O
- 400 gal 800 psi H₂O₂
- Hydraulic Control Valves



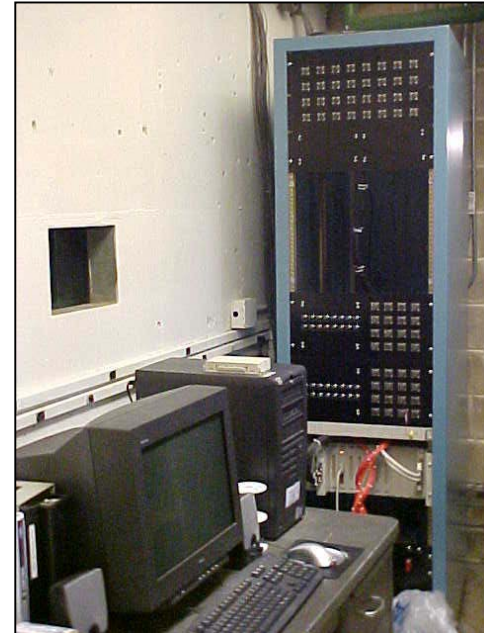
10,000 lbf Thrust Test Cell



- LabView 6.1-based DACS
- 10,000 lbf thrust measurement
- 64 channels pressure
- 96 channels thermocouples
- 18 channels analog control
- 32 channels on/off control

Control System Operation

- Data System Located Adjacent to Test Cell
- Operation Remoted to Control Room (KVM Extender) for Testing
- Video Recorded Directly to DVD



Test Cells



- 18” Thick Reinforced Concrete Test Cell Walls
- High Flow Capacity Test Cell Exhaust Fans
- Heated High Pressure Air Plumbed to Both Cells
- Walled Containment Area

Injector Characterization Scaling Approach

- Study Objectives

- Steady state and dynamic characterization of ORSC MC injector elements

- Approach

- Investigate full-scale elements at realistic operating conditions

- No film cooling (if possible)

- Evaluate different injector design configurations

- Couple with analysis

- Measurements

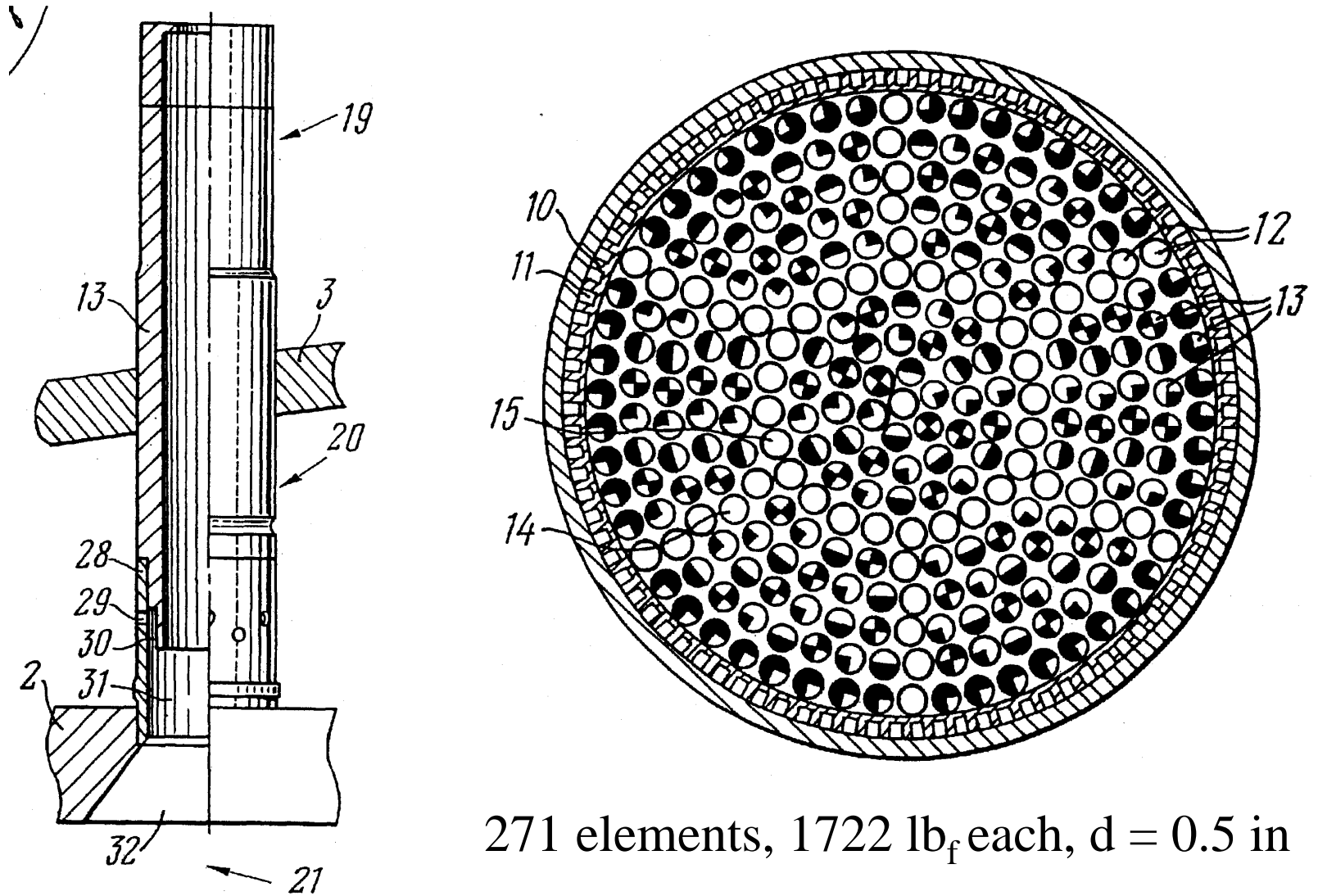
- Energy release profile from axial pressure gradient

- Injector face and chamber wall thermal environments

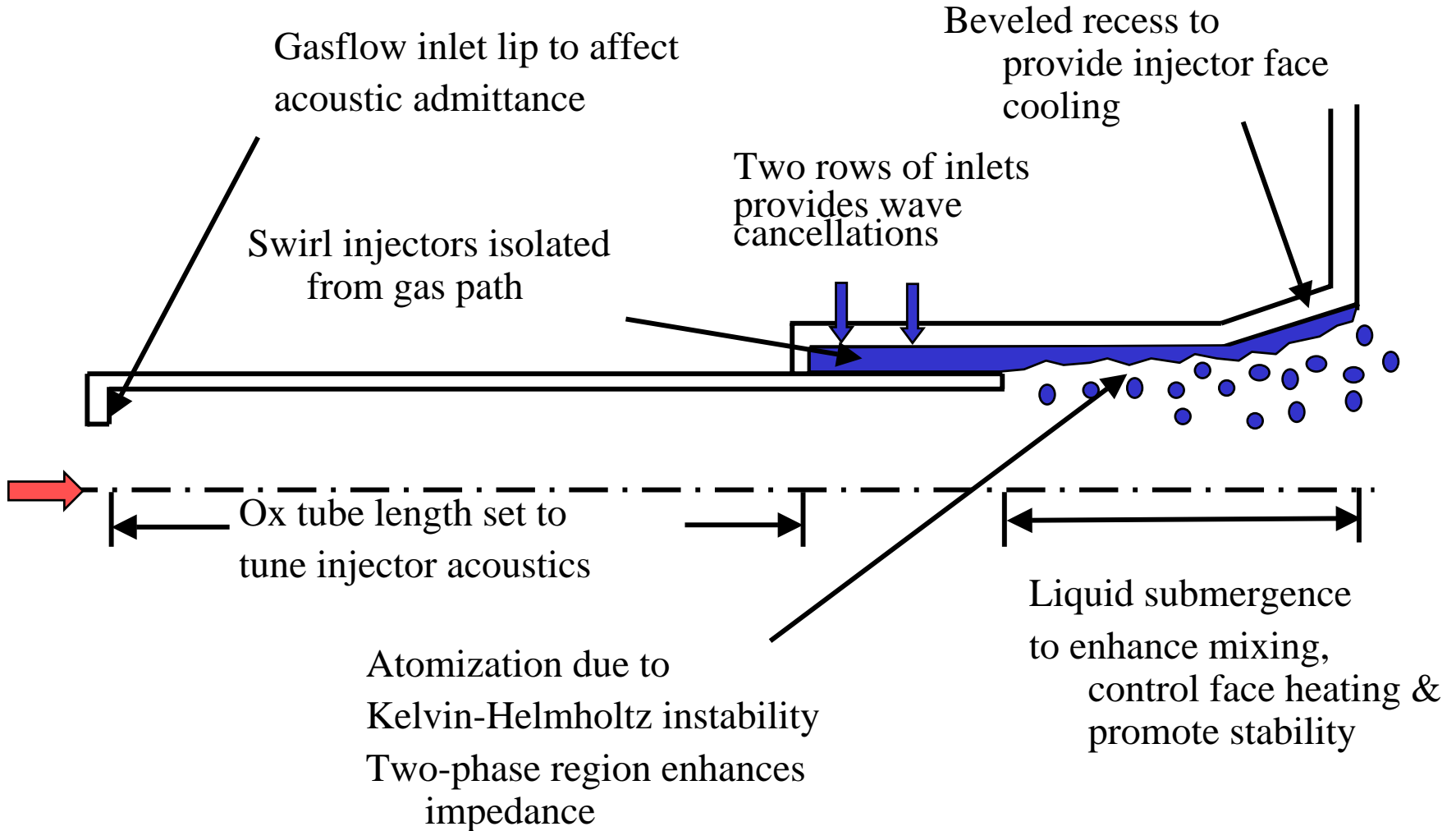
- Plume signature with IR tomography

- Manifold, injector and chamber p'

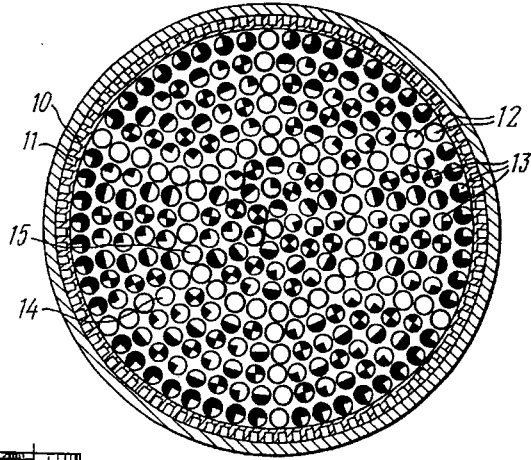
ORSC Main Combustor Components



Principle Design Features

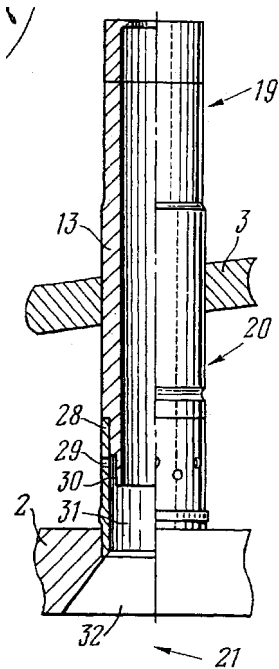


Single Element Sizing Exercise



Approach

- use full scale F/element ($1722 \text{ lb}_{\text{fvac}}$)
 - ➔ $\text{mox} = 3.6 \text{ lb/s}$, $\text{mf} = 1.2 \text{ lb/s}$
- test at 'full' P_c (2250 psia)
 - ➔ $A_t = 0.39 \text{ in}^2$, $d_t = 0.70 \text{ in}$
- match injection pressure drops (10%)
 - ➔ $d_{\text{inj, ox}} = 0.43 \text{ in}$, $d_{\text{inj}} = 0.57 \text{ in}$



Possible scaling methods:

Contraction ratio (1.61) ➔ $d_c = 0.89 \text{ in}$

Element to chamber area ratio (0.30) ➔ $d_c = 1.04 \text{ in}$

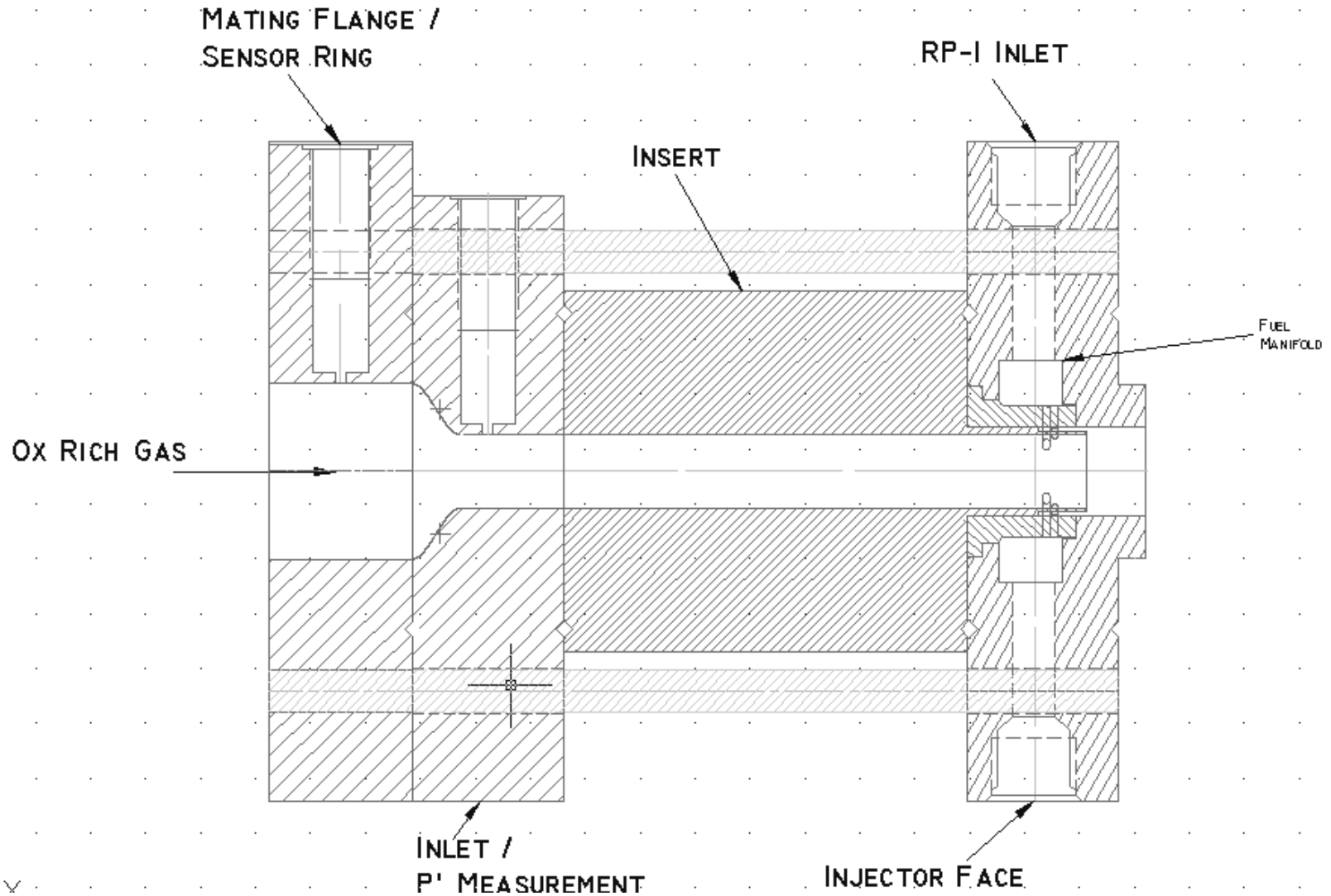
Element-element spacing ($0.60d$) ➔ $d_c = 0.91 \text{ in}$

Element-wall spacing ($0.60d$?) ➔ $d_c = 0.91 \text{ in}$

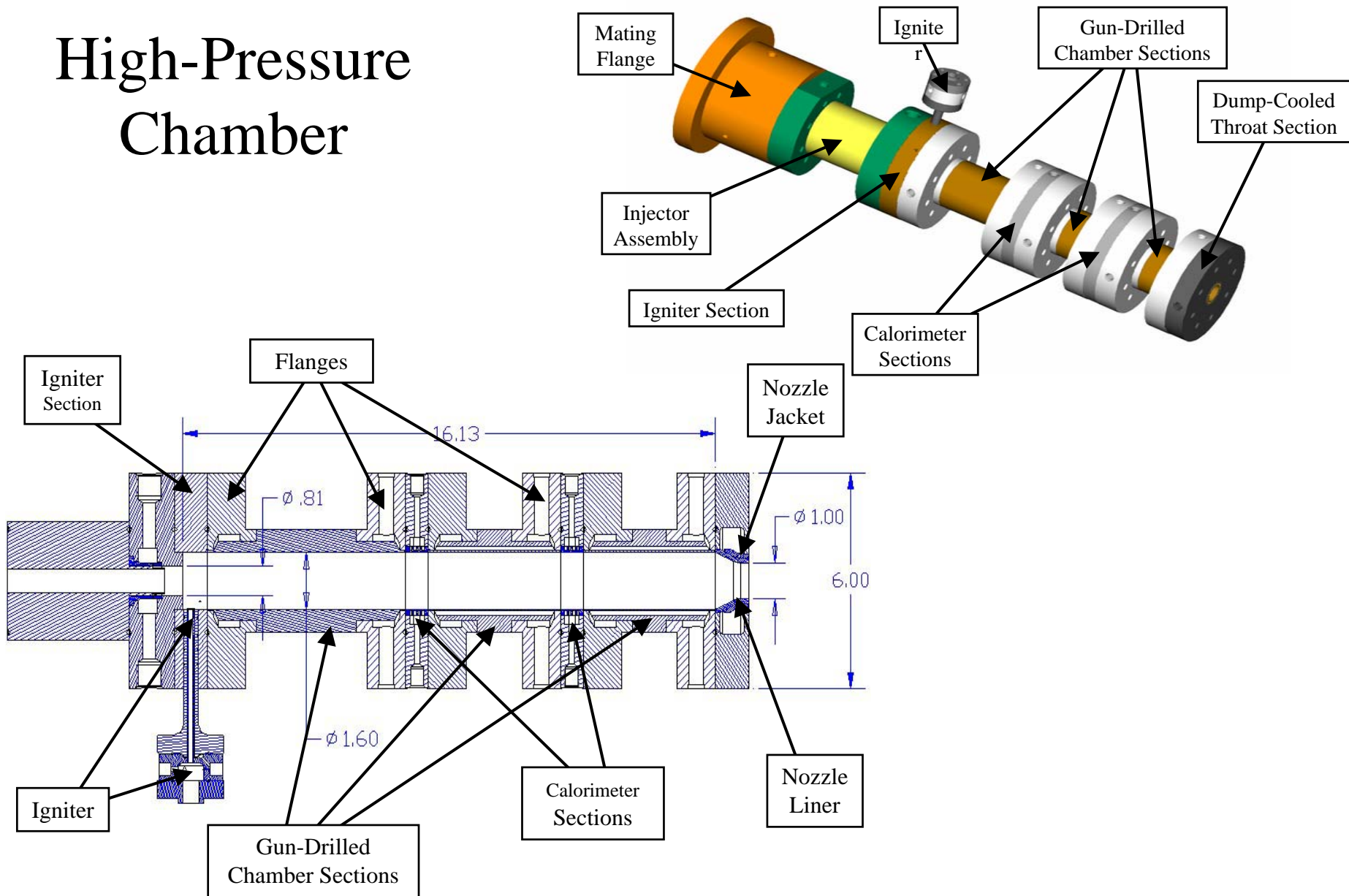
Element area (0.65 in^2) ➔ $d_c = 0.91 \text{ in}$

Chamber length based on $L^* \sim 30 \text{ in}$ (??)

Baseline Injector Design

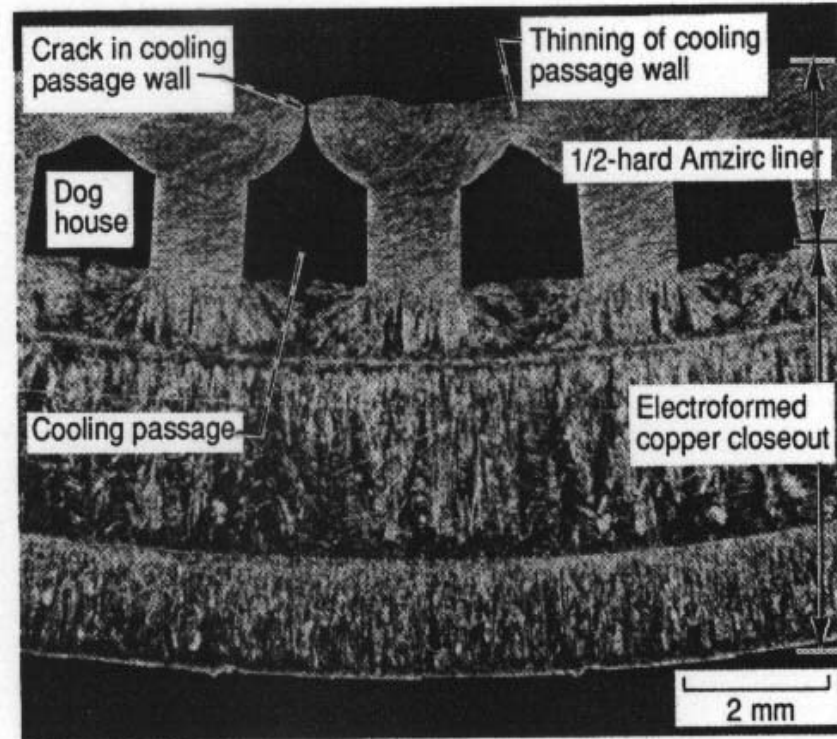


High-Pressure Chamber



Life Prediction - Background

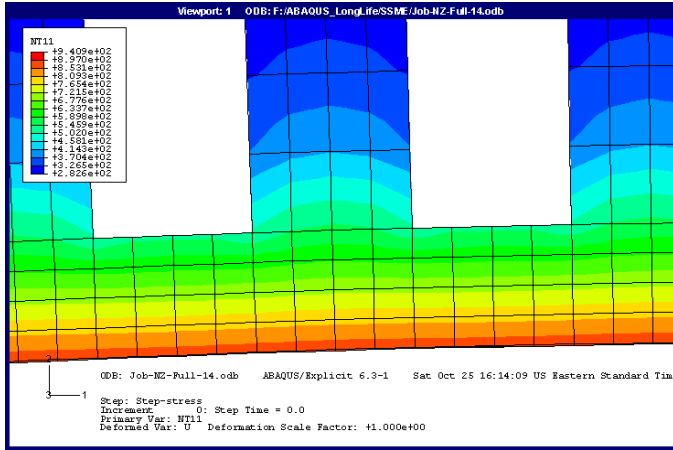
- Rocket combustor liner such as SSME operated at high temperature (6000F) and pressure (3000 psi) ranges as well as extreme heat flux (80 Btu/in²-s) requires active cooling devices to prevent material failure.
- Combustor liner experiences high thermal structural stress (~100 MPa) during mission profile (SSME 8 min)
- Experiments by Quentmeyer and Jankovsky showed bulging and thinning of liner due to cyclic loading
- Kasper and Porowski developed analytical life prediction methods using simple fatigue and creep model
- Robinson, Arnold and Freed developed visco-plastic model for fatigue-creep interaction phenomena which is believed to be a main failure mechanism



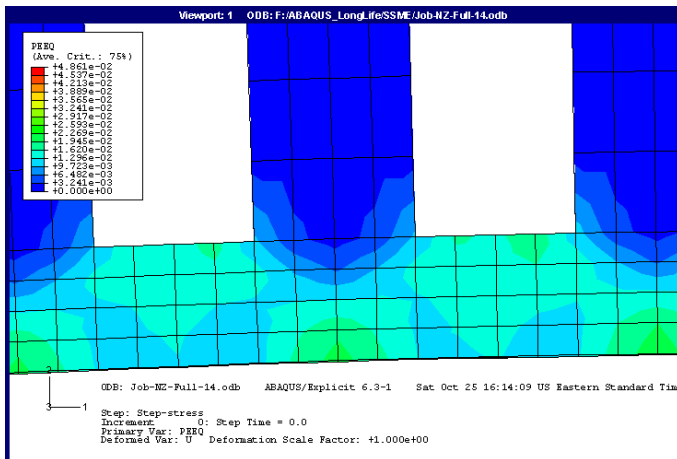
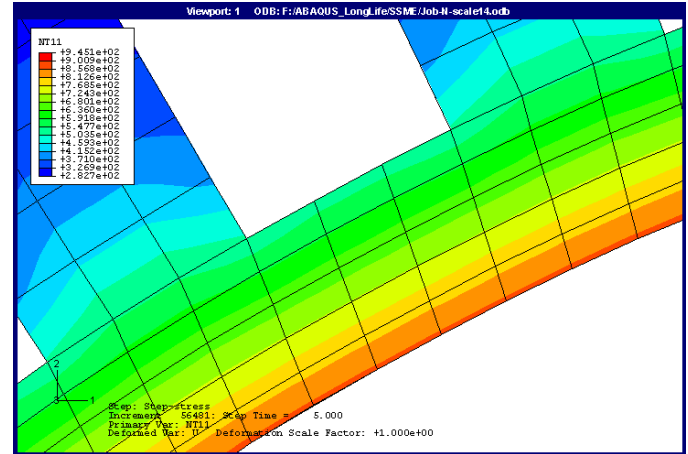
Typical failure mode of combustor liner at throat so called “dog house effect” per Quentmeyer

Full Scale – Subscale Life Comparison

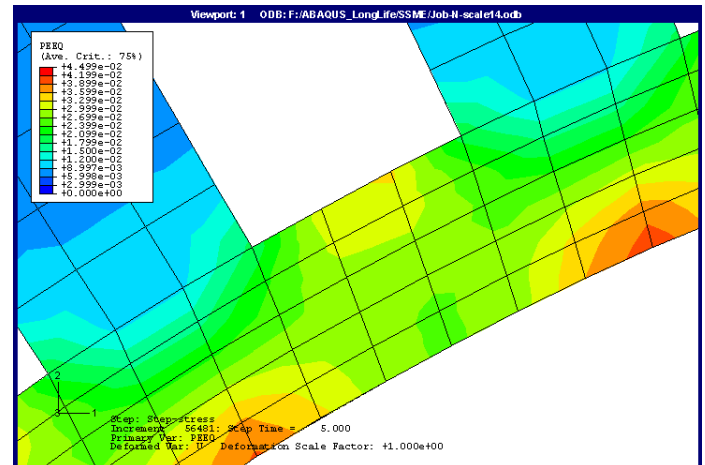
– $P_c = 3300$ psi, $T_c = 3500$ K



T



ε



Full scale engine

Strain_max = 2.4

Life = 120

1/10 scale model

Strain_max = 3.94

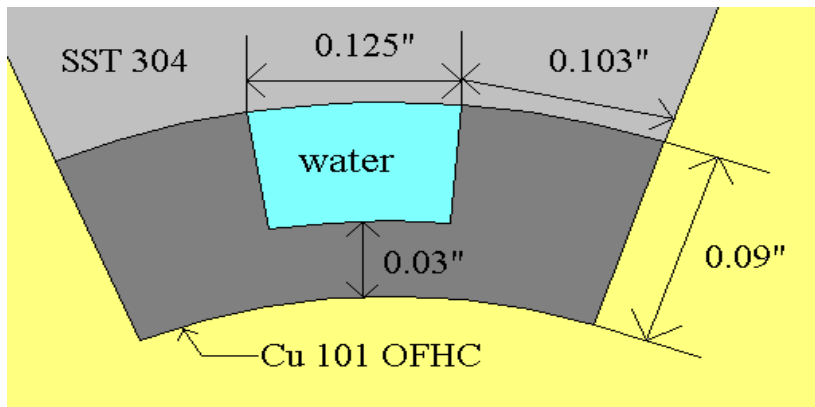
Life = 48

Approach

- Develop DBT course with life prediction as part of AAE curriculum
- Develop design requirements
 - Controlled hot-gas environments – use ‘pre-combustor’
 - Creep-fatigue interaction failure of cooled liner
 - Failure within reasonable number of cycles
- Life prediction analysis using conventional methods
 - Chemical equilibrium in pre-combustor
 - One-dimensional heat transfer analysis for initial design
 - critical heat flux and cooling requirements, duty cycle
 - FEM for stress and plastic strain
 - Strain-life curves for cycle life
 - More advanced life modeling by graduate student following project
- Cyclic testing of test article
 - Ten cycles per test
 - Validation of cooling analysis
 - Regular inspection
- Test-to-failure

Combustor Design Parameters

- Top level requirements
 - Less than 200 life cycle
 - Test should produce verifiable results
 - Liner has no melting prior to the LCF failure
 - All parts had to be manufactured in ASL at Purdue
- Under these requirements, the coolant pressure, flow rate and cooling channel aspect ratio (0.5) were determined.



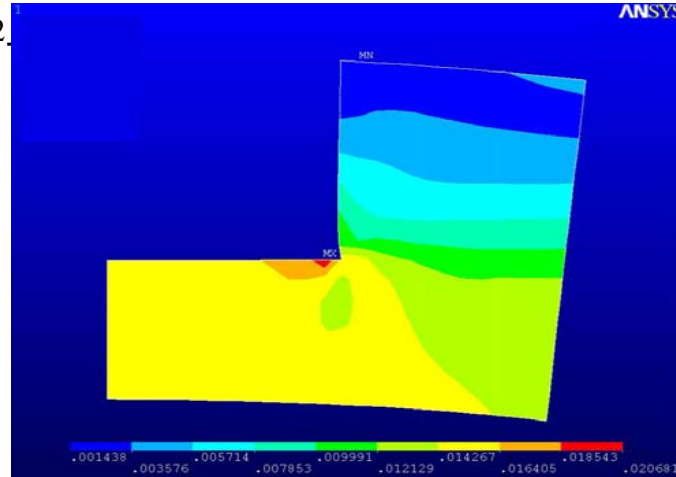
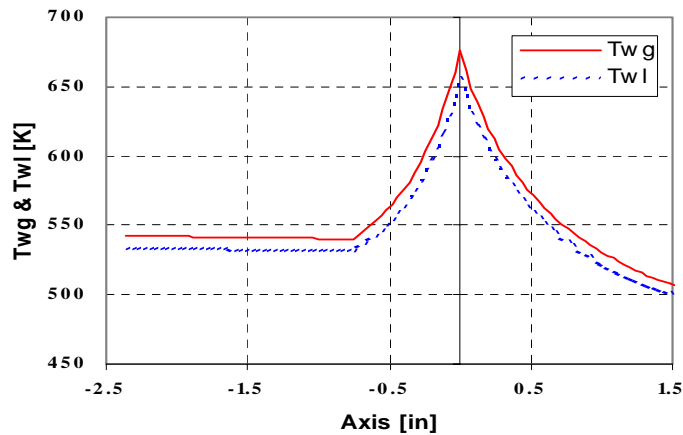
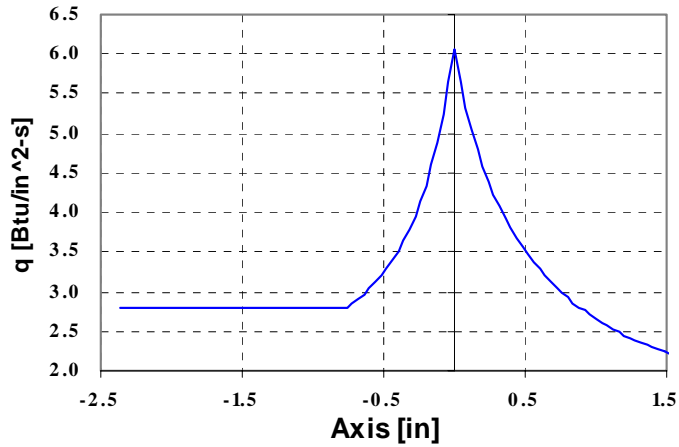
| Parameter | Value |
|---------------------------------------|--|
| Propellant | 90% H ₂ O ₂ + JP-8 |
| Propellant mixture ratio (O/F) | 4.0 |
| Propellant flow rate | 1.25 lb/s |
| Chamber pressure (P _c) | 200 psia |
| Chamber temperature (T _c) | 3440 °F |
| Characteristic velocity (C*) | 4961 ft/s |
| Throat area (A _t) | 0.915 in ² |
| Characteristic length (L*) | 70 |
| Test liner diameter | 2.0 in |
| Test liner length | 5.0 in |
| No. of cooling channel | 30 |
| P _{coolant} | 110 psi |
| M _{dot} _{coolant} | 0.8 lb/s |

Table 1 : Combustor design parameters

Thermal Structural Prediction

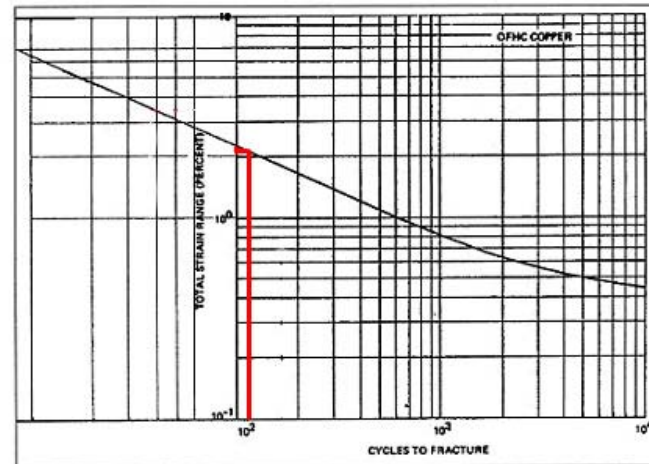
Thermal analysis

- Burn out heat flux --- 6.54 Btu/in²
- Max wall temp --- 670 K



Total strain predicted by ANSYS around rectangular cooling channel.

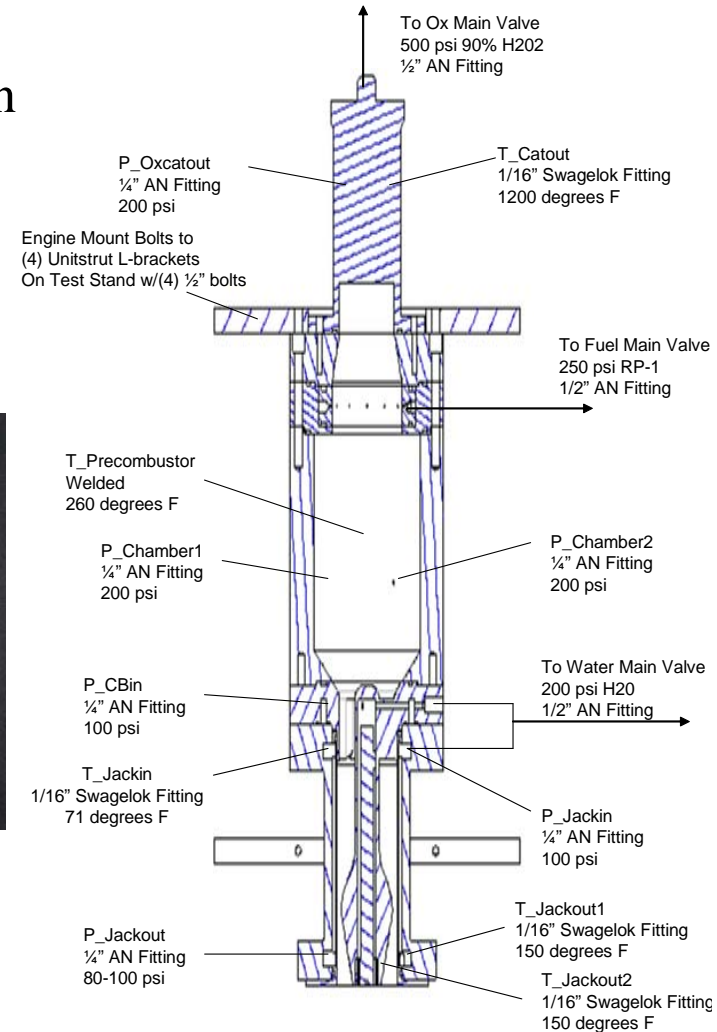
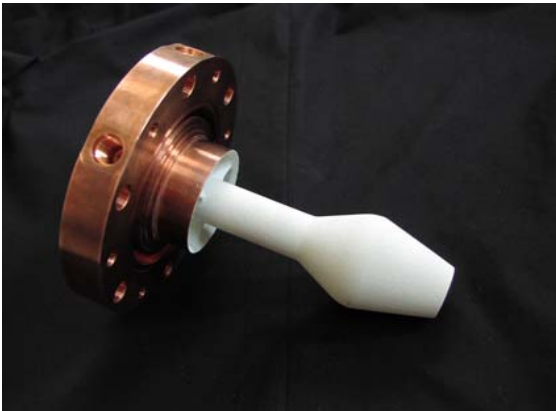
- Total strain --- 2.0 %
- Life expected --- 115 cycles



Strain-life curve for OFHC at 810 K from NASA CR-134806, 1975

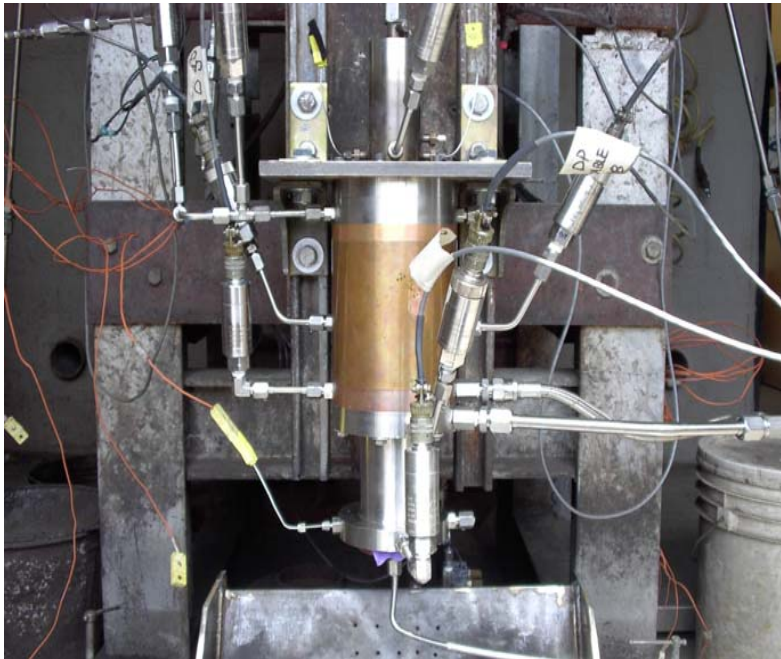
Test Article

- Catalyst bed for decomposing H_2O_2
- Heat sink dump combustor for hot gas generation
- Chamber liner --- water cooling
- Center body --- water cooling with TBC (0.01" thick)



Testing

- Tests were conducted in the APCL at Purdue University
- Propellant flow timing sequence was automatically controlled by pneumatically actuated valve with LABVIEW system



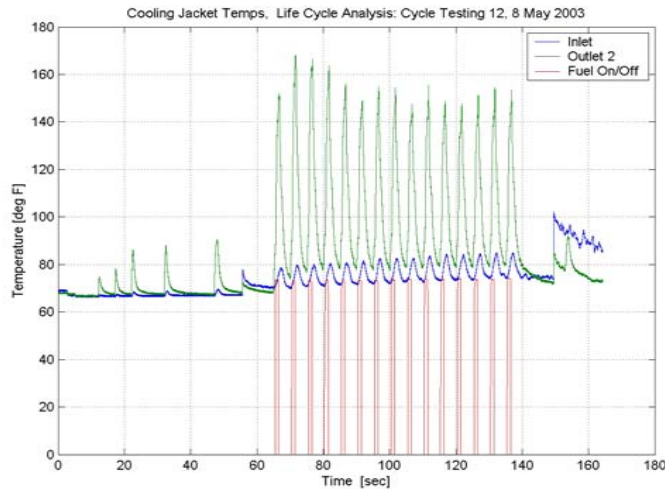
Test article assembly on test stand



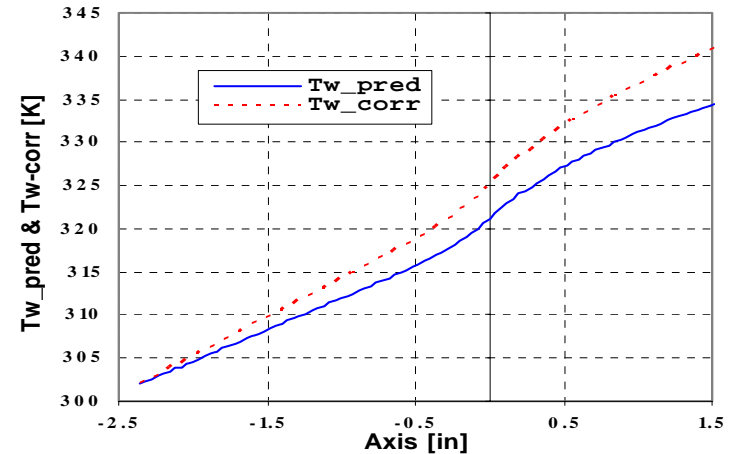
Cyclic test

Test Results

- Chamber pressure, C^* efficiency, propellant mass flow rate, coolant temperature and pressure were measured and calculated
- Data reduction was performed using in-house code written by students using MATLAB
- Validation procedure
 - Measure coolant ΔT , wall thinning rate
 - $2.15E-5$ in/cycle ($0.032'' \rightarrow 0.029''$)
 - Verify 1D thermal model
 - Compute updated thermo-structural environment
 - Make life prediction



Coolant temperature



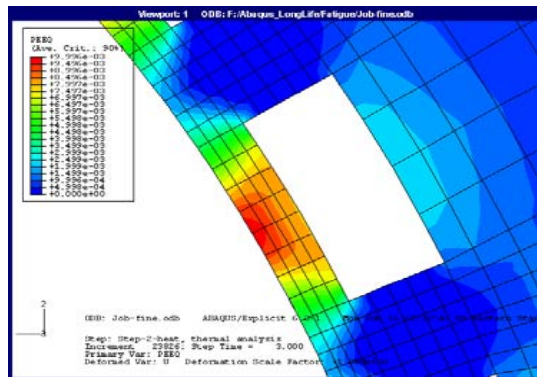
Predicted and measured coolant temperature
 $\Delta T = 4.0K$ at throat



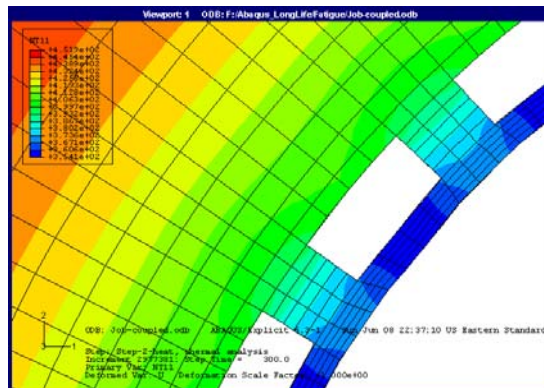
Discoloration and deformation at
90 cycles ($1.5'' \times 0.6''$)

Updated Structural Analysis

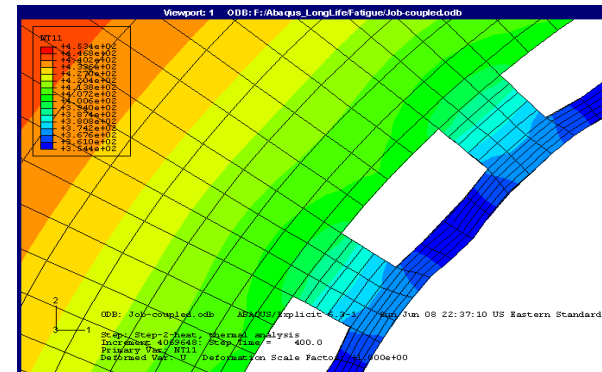
- Simulation of temperature, strain and deformation (bulging, thinning) using ABAQUS explicit module
- Maximum strain : 1.2 % at middle of ligament
- Only bulging of ligament was simulated



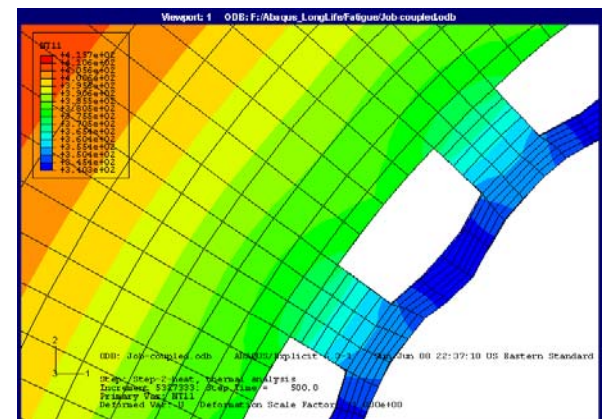
Plastic strain distribution



Deformation after 60 cycle



Deformation after 80 cycle



Deformation after 100 cycle

Summary and Conclusions

- Small-scale rocket combustor was designed and tested to verify life prediction models for low cycle fatigue and fatigue-creep interaction.
- Several life prediction methods were applied to predict combustor life and were compared with test results.
- Correlation data used to improve predictions.
- Improvements would include fixing the liner lands to the structural jacket, and testing at more severe conditions.

| Prediction method | Estimated life cycle | Determined life cycle by experiment |
|------------------------------|----------------------|-------------------------------------|
| Effective stress-strain | 115 | |
| ANSYS | 115 | |
| Porowski | 51 | 270 |
| Dai and Ray with Freed model | 260 | |
| ABAQUS | 320 | |

Comparison of life prediction with test

Summary and Conclusions

- 100's of cycle goal is very challenging and verification would be very expensive
 - Question of economic feasibility
- Improved life prediction methodology for expanding range of design and operational scenarios is needed
 - Probabilistic life prediction design analysis
 - Testing methodologies with *in situ* thermostructural response measurements
 - Environments definition
 - Improved material database and understanding of damage mechanisms

Acknowledgements

- Work sponsored under NAG8-1856, -1876, -1894
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- Students of AAE 590
- School of Aeronautics and Astronautics