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Local Heat Flux Measurements with Single Element Coaxial Injectors

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ABSTRACT

To support the mission for the NASA Vision for Space Exploration, the NASA Marshall Space Flight Center conducted a program in 2005 to improve the capability to predict local thermal compatibility and heat transfer in liquid propellant rocket engine combustion devices. The ultimate objective was to predict and hence reduce the local peak heat flux due to injector design, resulting in a significant improvement in overall engine reliability and durability. Such analyses are applicable to combustion devices in booster, upper stage, and in-space engines, as well as for small thrusters with few elements in the injector. In this program, single element and three-element injectors were hot-fire tested with liquid oxygen and ambient temperature gaseous hydrogen propellants at The Pennsylvania State University Cryogenic Combustor Laboratory from May to August 2005. Local heat fluxes were measured in a 1-inch internal diameter heat sink combustion chamber using Medtherm coaxial thermocouples and Gardon heat flux gauges. Injectors were tested with shear coaxial and swirl coaxial elements, including recessed, flush and scarfed oxidizer post configurations, and concentric and non-concentric fuel annuli. This paper includes general descriptions of the experimental hardware, instrumentation, and results of the hot-fire testing for three of the single element injectors – recessed-post shear coaxial with concentric fuel, flush-post swirl coaxial with concentric fuel, and scarfed-post swirl coaxial with concentric fuel. Detailed geometry and test results will be published elsewhere to provide well-defined data sets for injector development and model validation.



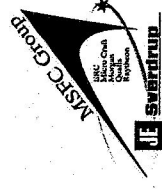
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Heat Transfer is Essential to Exploration Mission

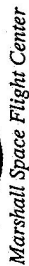
- In-space engines *must* be extremely reliable
 - Combustor compatibility and durability are critical factors in engine reliability
 - defined by *local* heat transfer, not bulk heat transfer
 - Current capability to analyze *local* heating effects from injector is insufficient and must be improved
- Some exploration engine cycles also *depend* on heat transfer to be operational
 - Expander and tap-off engine cycles use combustion chamber heat for turbine drive gas energy
- Past heat transfer design methods are not efficient
 - Previous engine development used mostly empirical methods and “test-fail-fix” design philosophy



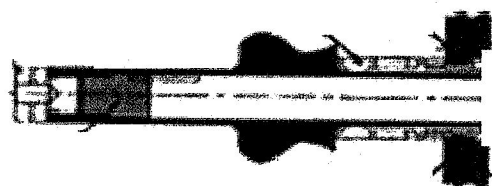
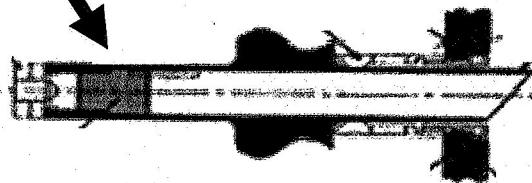
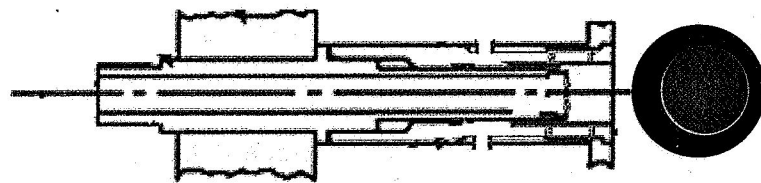
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MSFC Program Objective – Reduce *Local* Peak Heat Flux Due to Injector

- Improve local heat transfer analysis capability
 - Current capability to analyze local injector heating effects is largely one-dimensional and empirical
 - Improve computational fluid dynamic (CFD) model capability
 - Add features for three-dimensional flows, real fluids, and faster turnaround capability
 - Validate CFD model with highly-resolved small scale experiments
 - Multiple injection element types
 - Single-element and small multi-element
- Develop advanced injector designs to reduce local peak wall heat flux
 - Previous injectors developed by “test-fail-fix” were not optimized
 - Design, fabricate, and test advanced elements in highly-resolved small scale experiments



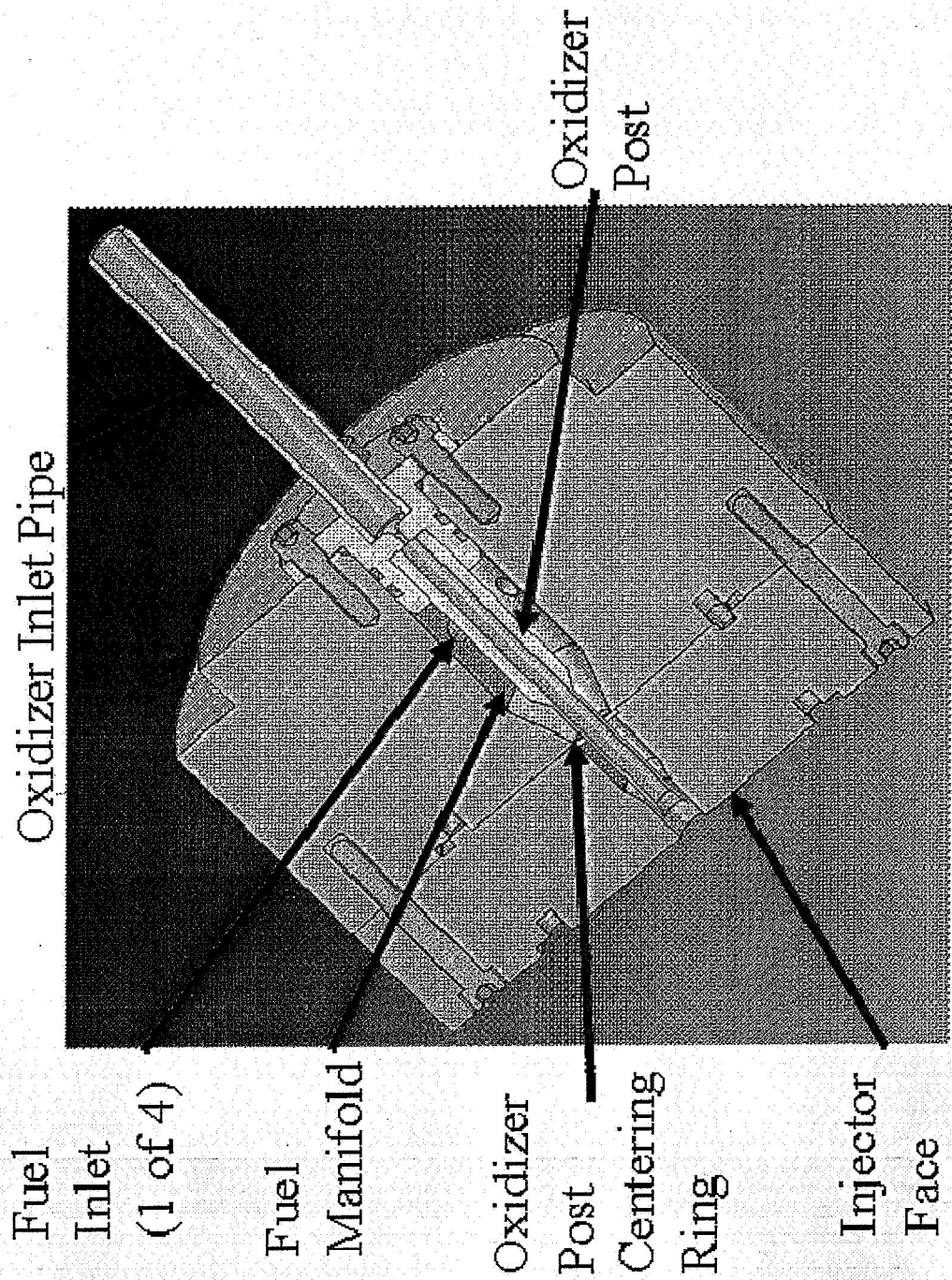
1. Baseline Shear Coax
(concentric fuel)
2. Off-Set Shear Coax
(non-concentric fuel)
3. Flush Tangential Swirl
4. Scarfed Tangential Swirl Coax





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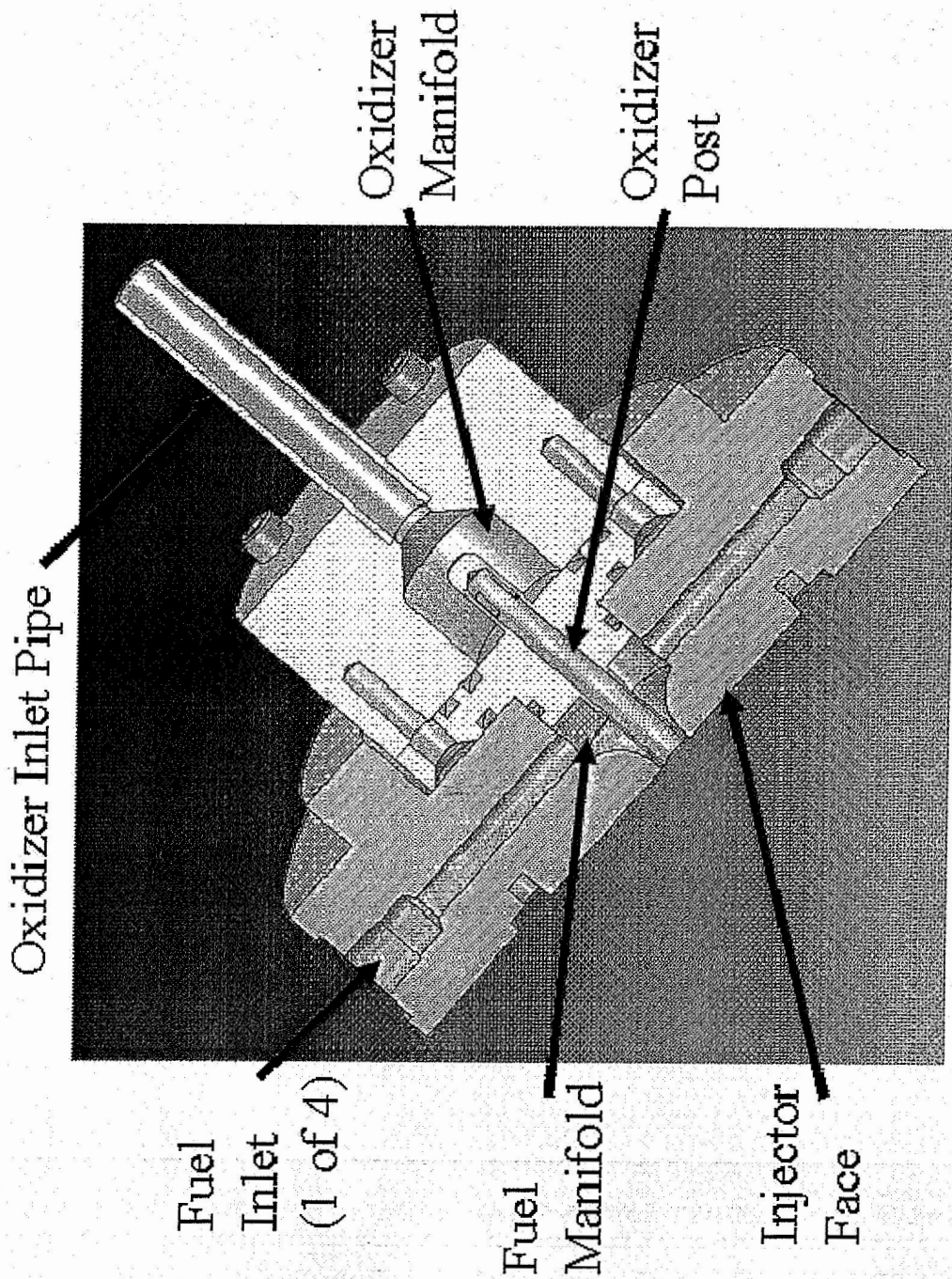
Single Element Shear Coax





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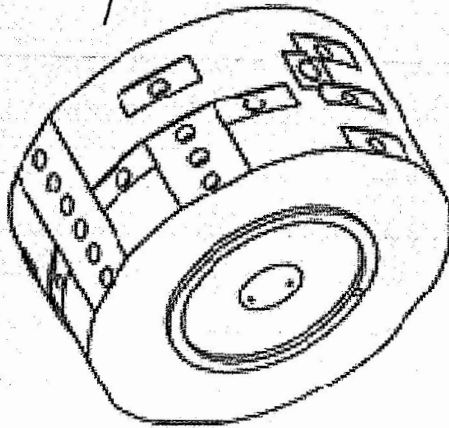
Single Element Swirl Coax



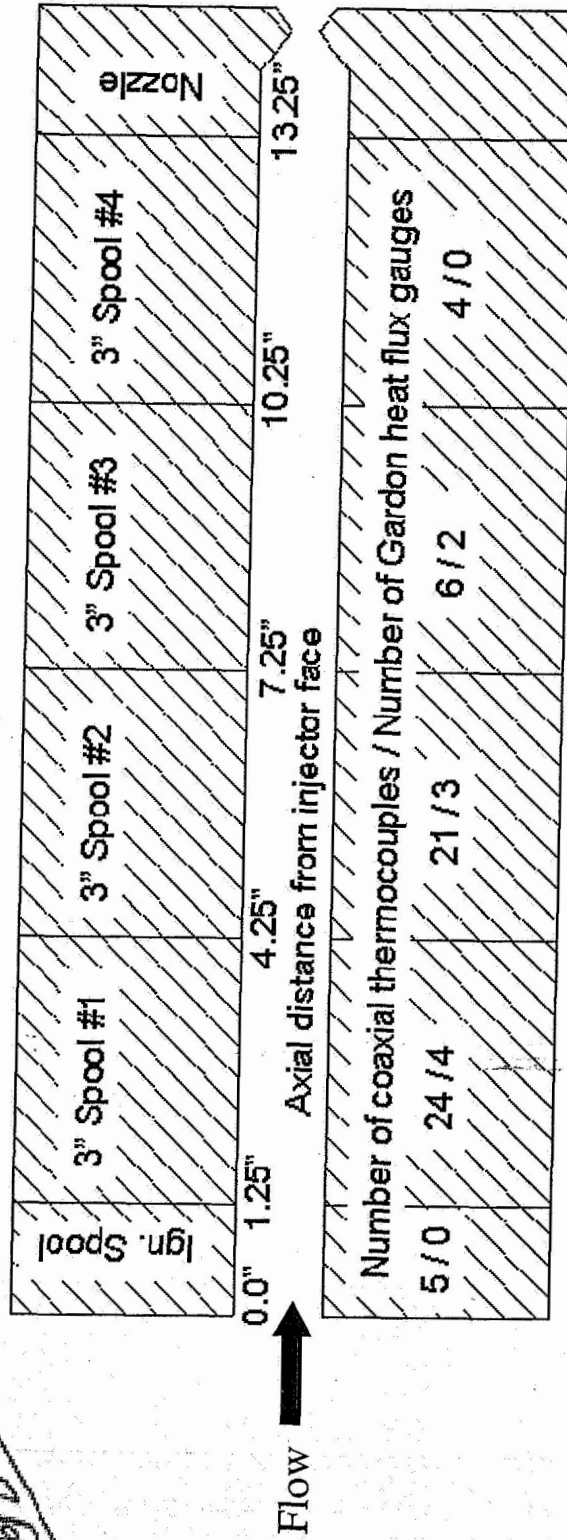


Compatibility/Heat Transfer Combustion Chamber

- Modular chamber with multiple spools
- 1-inch ID, 6-inch OD



Individual
Chamber
Spool

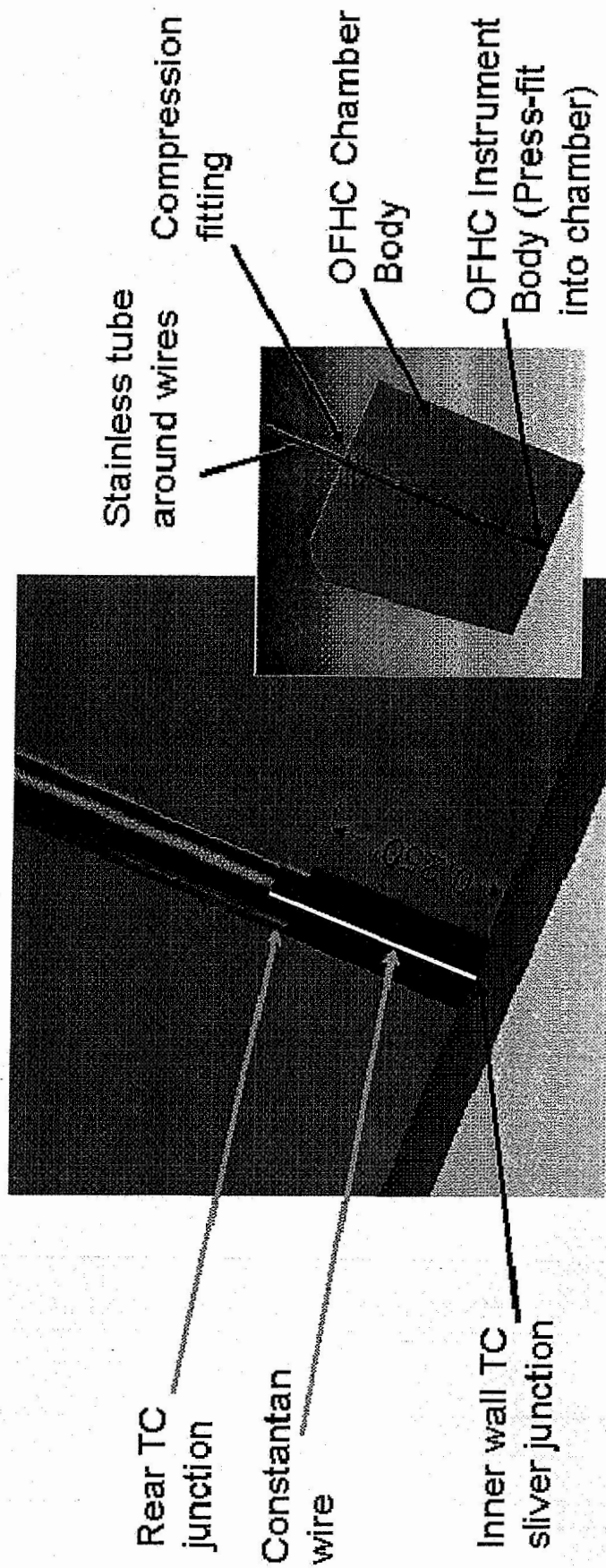


Layout of Chamber Spools with Instrumentation



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Medtherm Coaxial Thermocouple

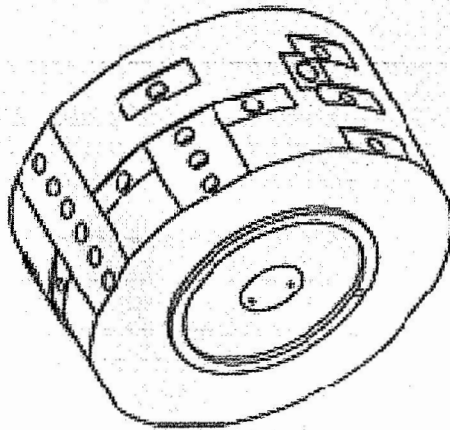




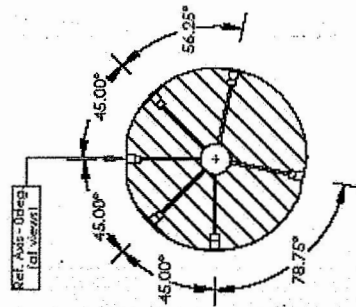
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Examples of Coaxial Thermocouple Layouts at Different Axial Locations

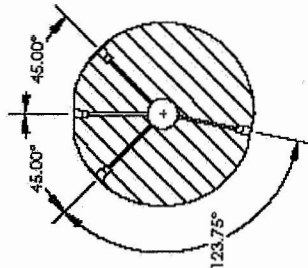
- Up to 10 sections per spool



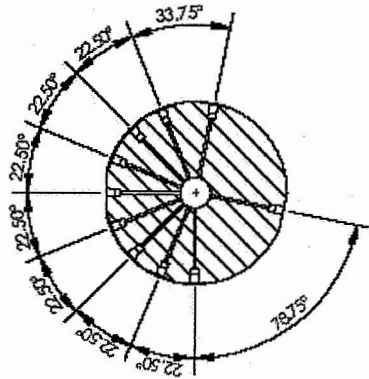
Individual
Chamber
Spool



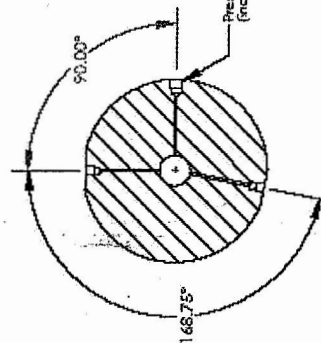
SECTION A-A



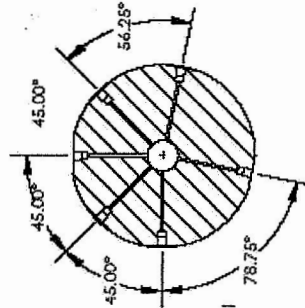
SECTION B-B



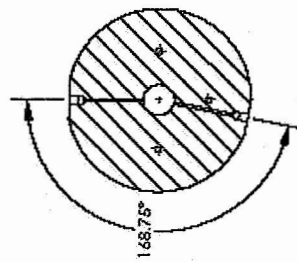
SECTION C-C



SECTION D-D



SECTION E-E

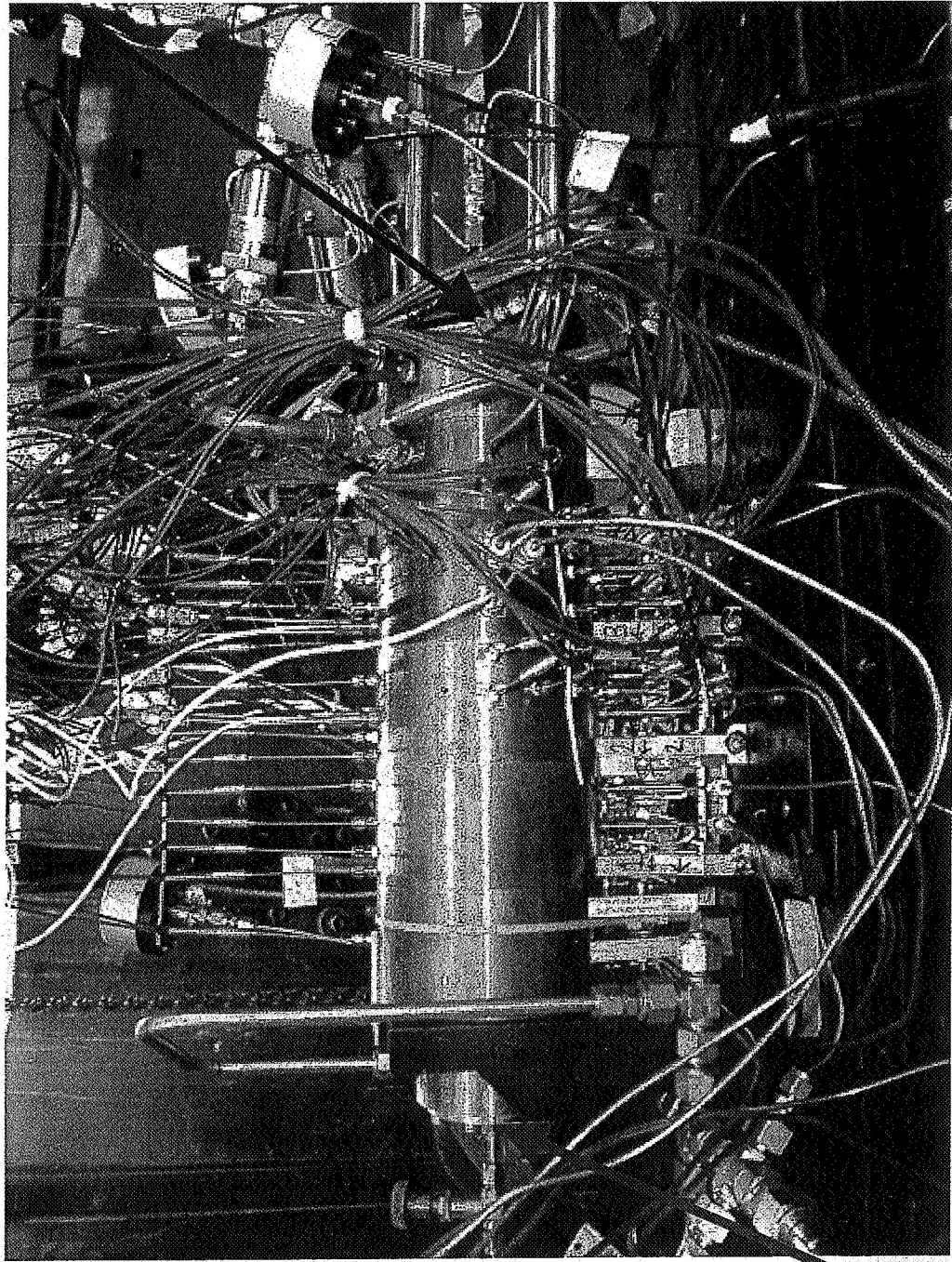
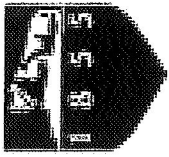


SECTION F-F



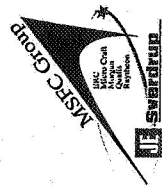
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Compatibility/Heat Transfer Test Rig at The Pennsylvania State University



Injector

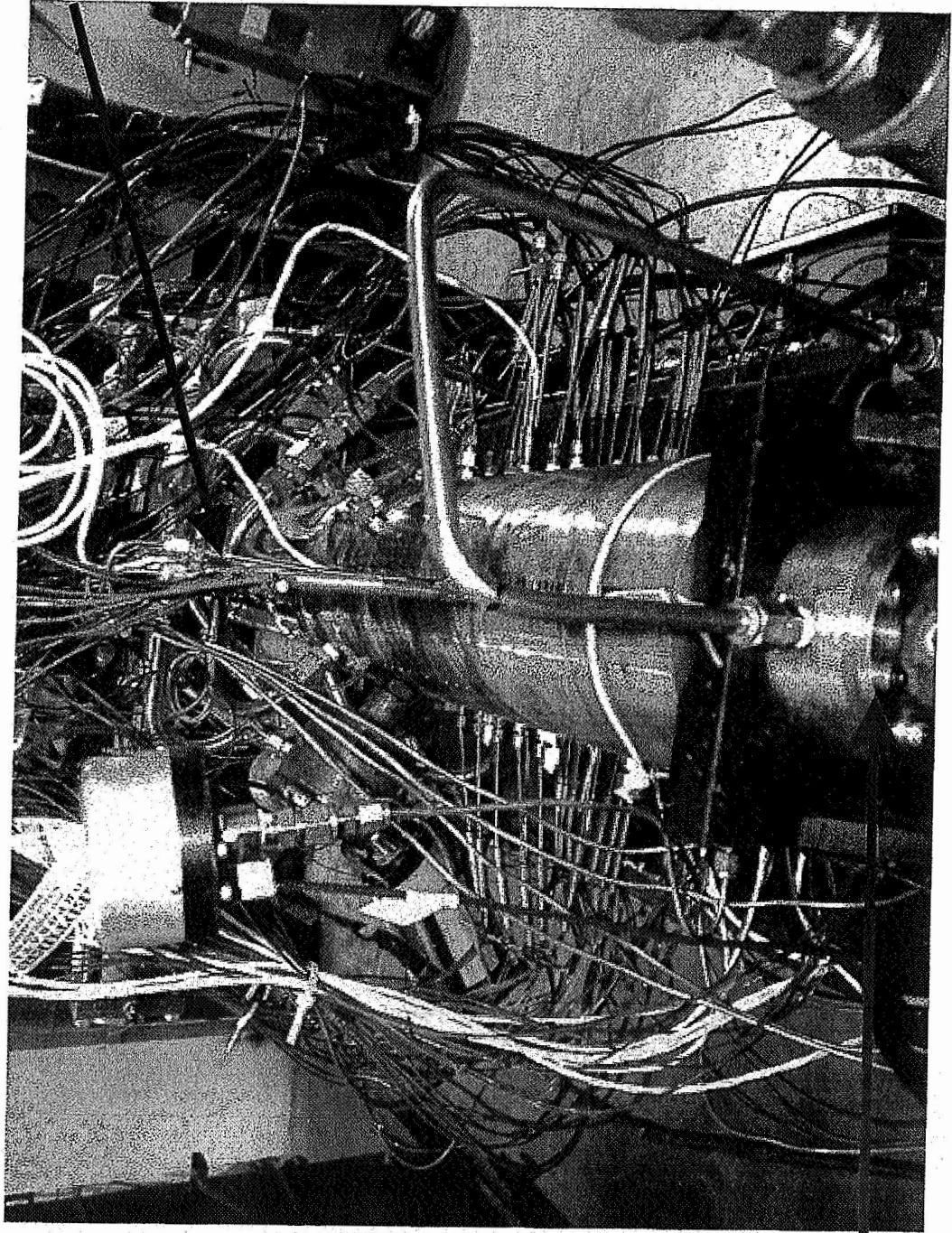
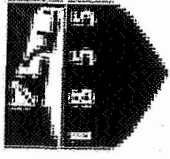
Nozzle





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Compatibility/Heat Transfer Test Rig at The Pennsylvania State University



Injector

Nozzle

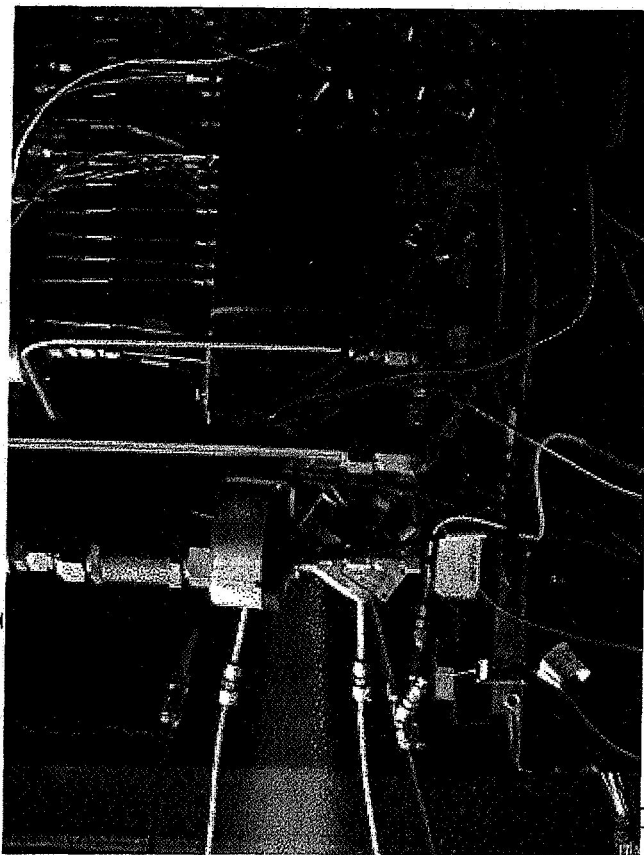
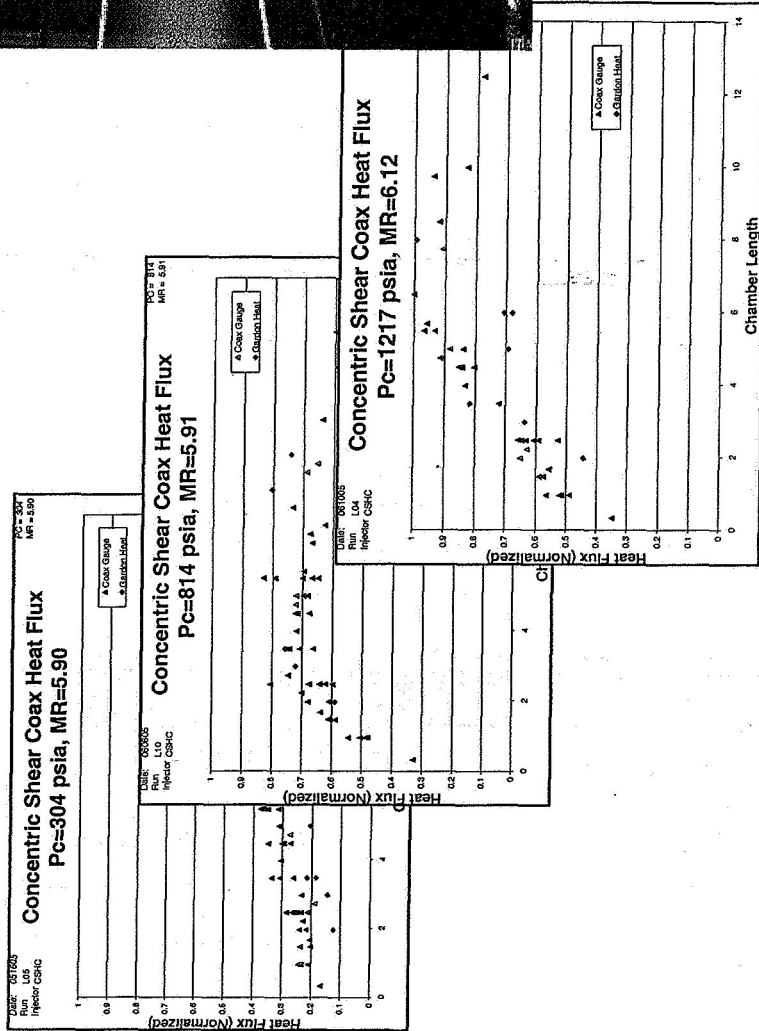




Over 100 Tests Completed at Penn State

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- 109 tests completed
- 10 injector configurations tested
- Chamber pressures varied from 300 – 1200 psia
- Mixture ratio varied from 5 to 6.5





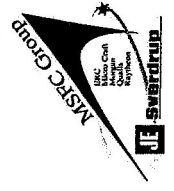
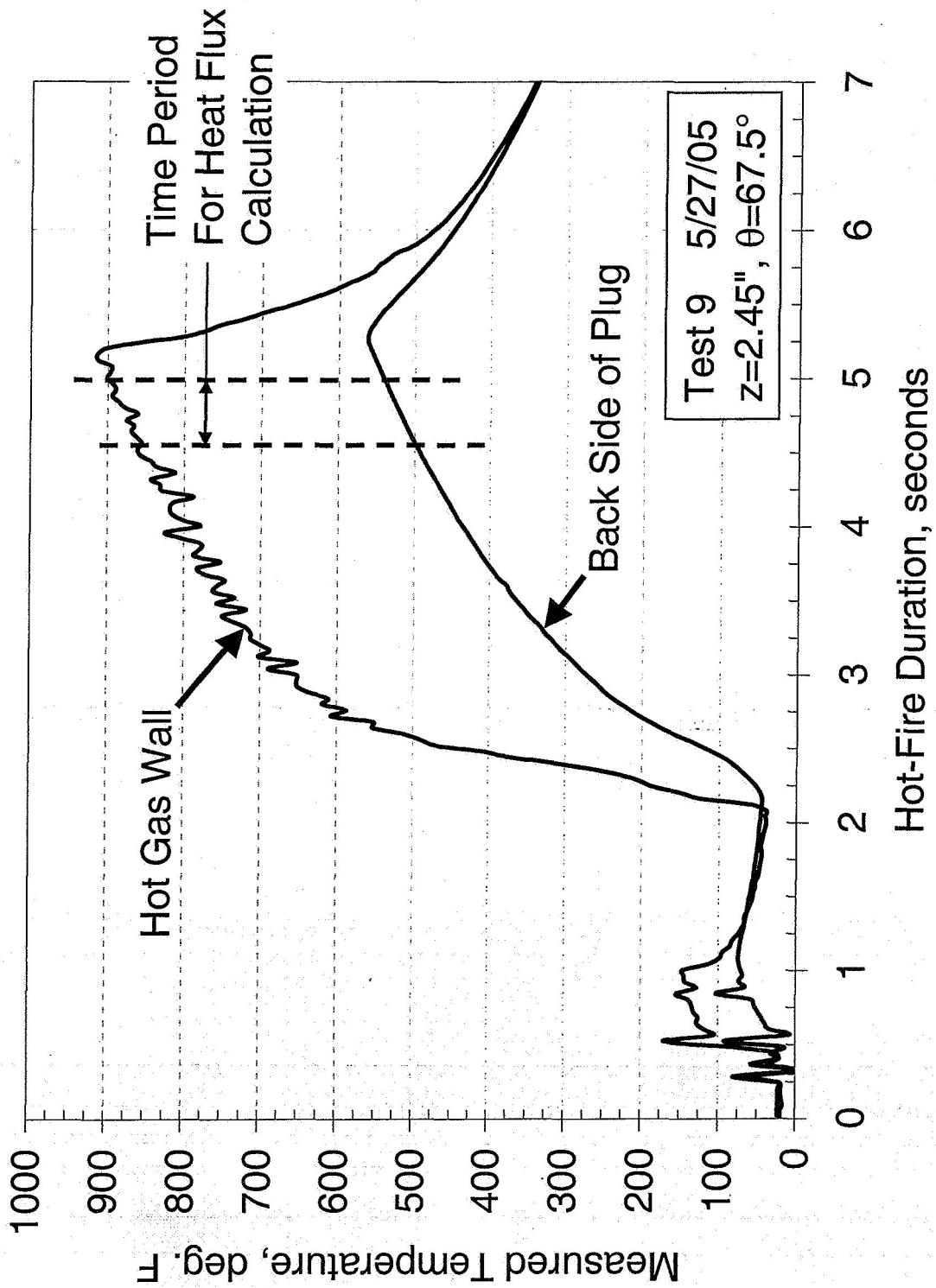
Evaluation of Test Data Validity

- Uncertainty of heat flux from Medtherm coaxial thermocouple
 - Previously evaluated at the PSU CCL for 2003 gas/gas testing
 - Calculated uncertainty ~ 0.6 %
- Accuracy and repeatability
 - Compared normalized heat flux from different gauges in same location after injector was rotated
 - Analysis of many tests from 4 injectors and 3 mixture ratios
 - Includes run-to-run and gauge-to-gauge variability
 - Average deviation calculated ~ 3%
- Effect of contact of press-fit plug with the bulk chamber
 - Effect varies as a function of test duration
 - Raw data examined and averaged summary period selected to exclude effect
- Effect of plug recessing or protruding into chamber
 - Specific locations noted; evaluation in progress with CFD analyses
- Effect of selection of summary period with variable test durations
 - Heat flux naturally biased lower the later the data collected due to wall temp
 - If wall temperature not included, added variability ~ 1%



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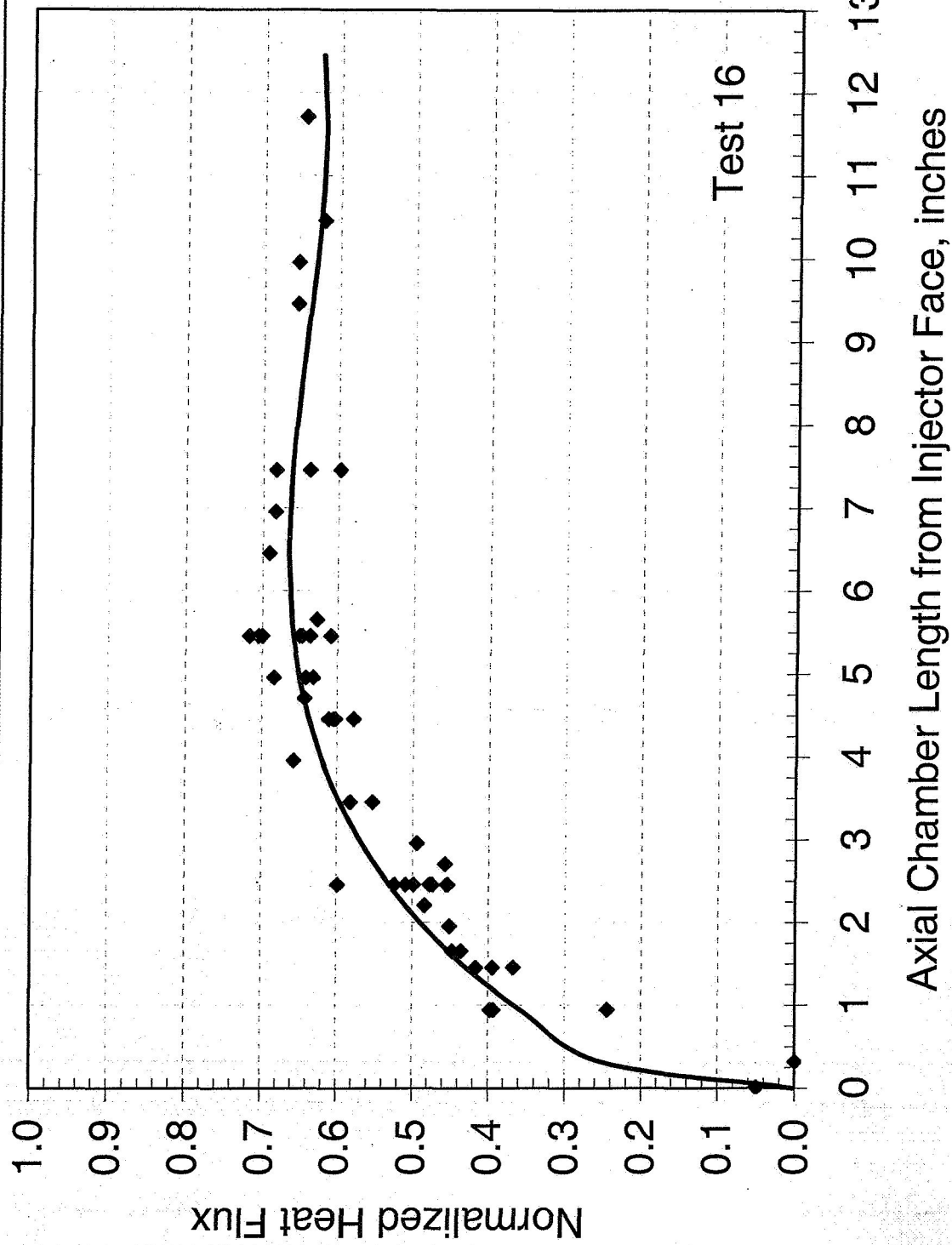
Heat Flux Calculated From Coax Thermocouples





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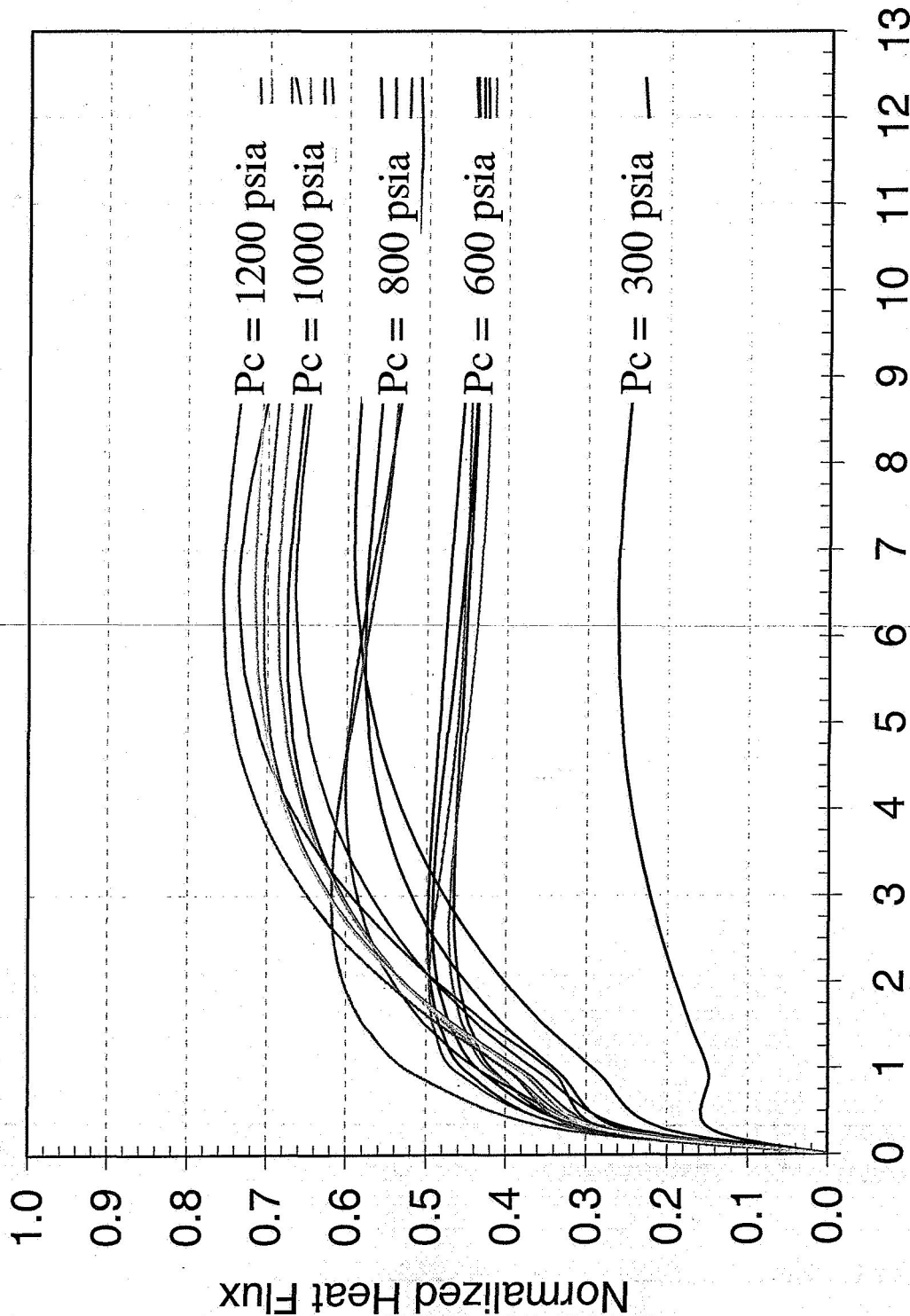
Measured Heat Flux Data Fit to High-order Polynomial Function





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Heat Flux Data for Concentric Shear Coax

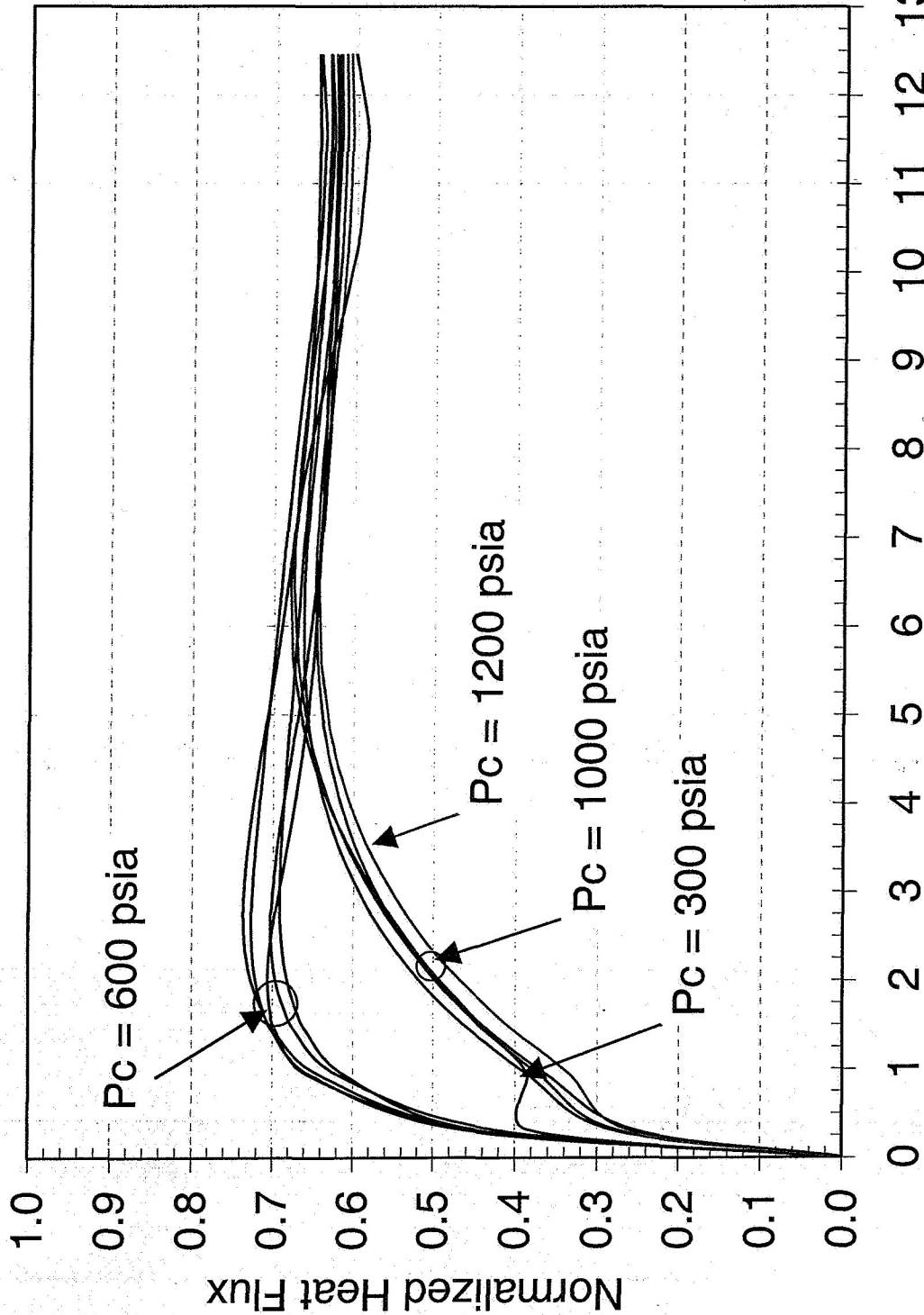


Axial Chamber Length from Injector Face, inches



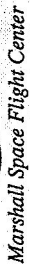
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Concentric Shear Coax Heat Flux Collapses to Two Separate Groups with $P_c^{0.8}$



Axial Chamber Length from Injector Face, inches

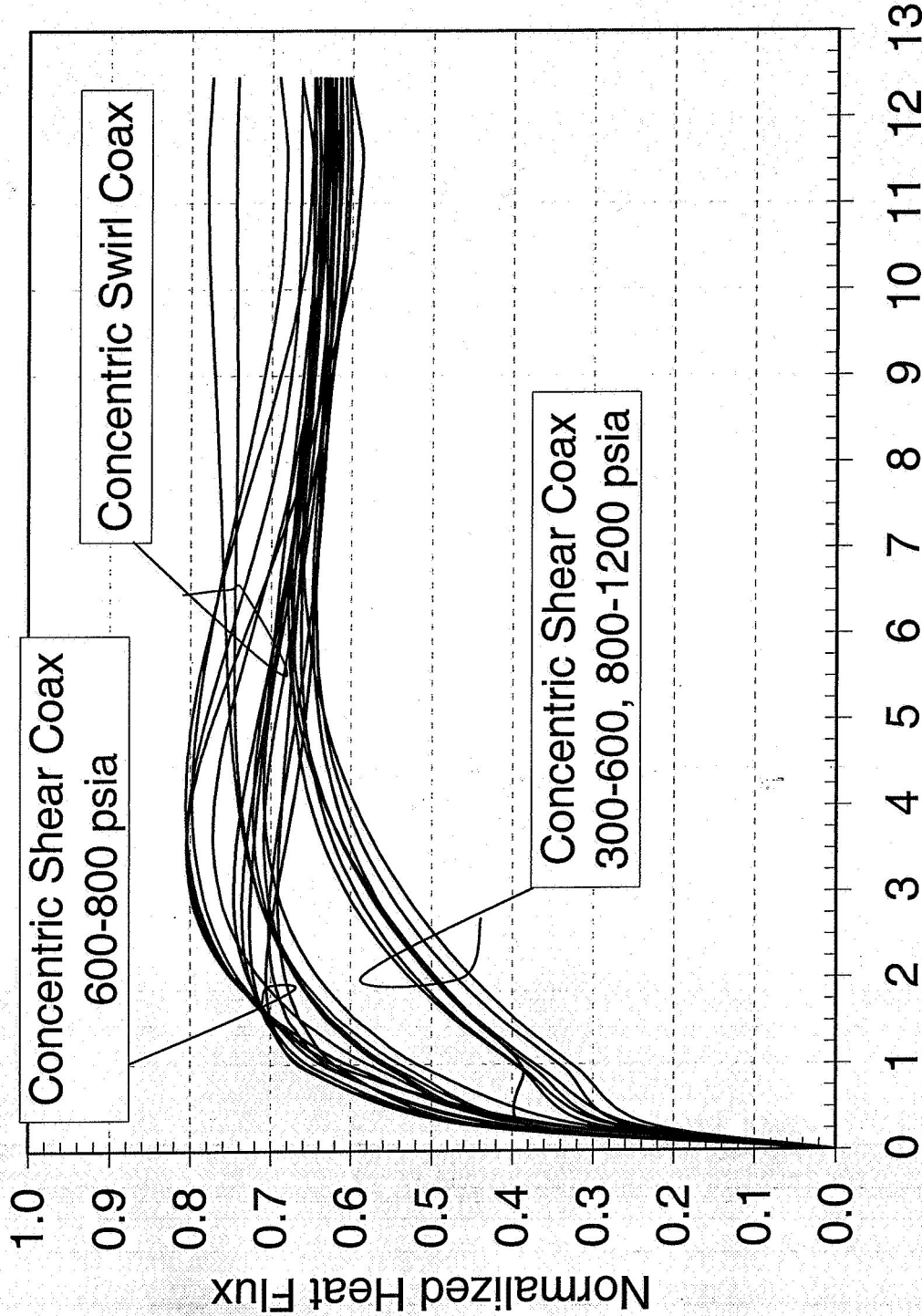




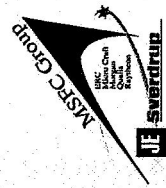


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Concentric Swirl Coax Versus Concentric Shear Coax



Axial Chamber Length from Injector Face, inches



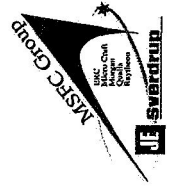
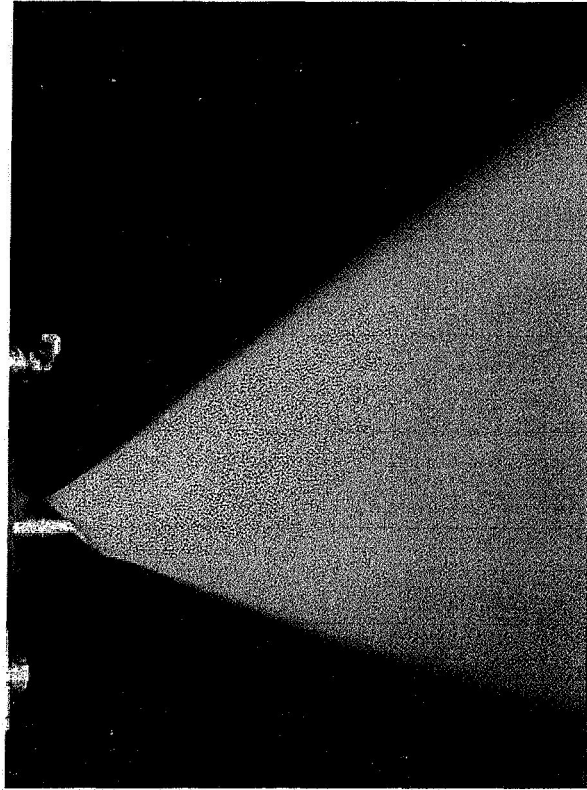
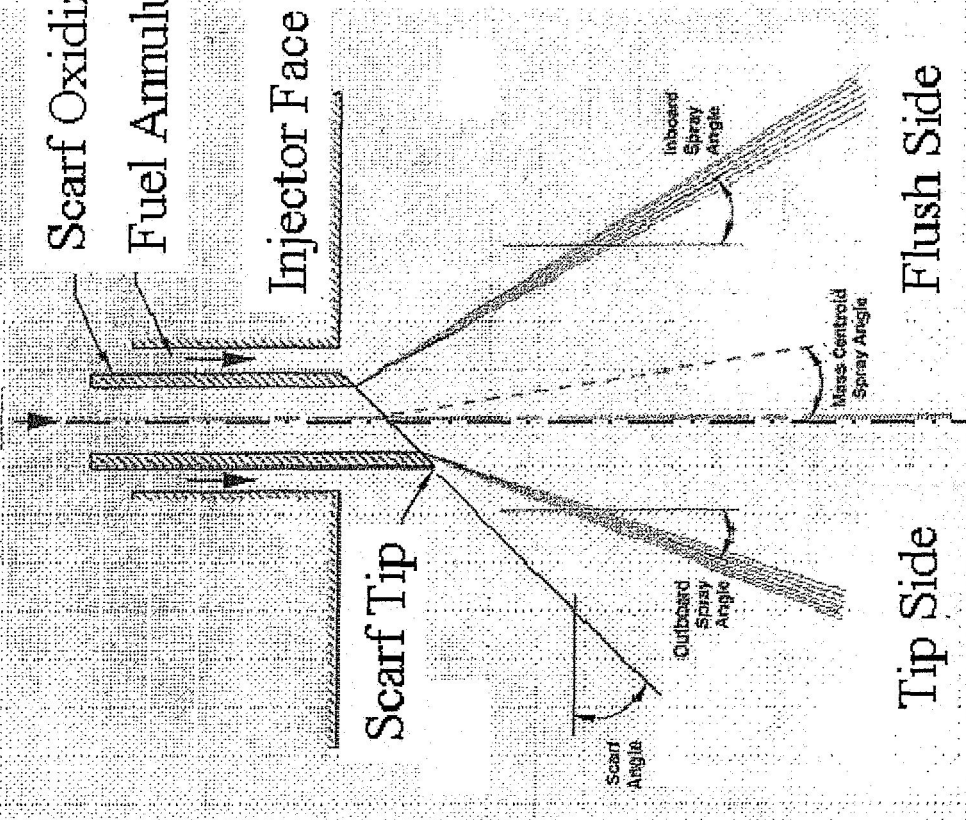


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Scarf Swirl Coax Nomenclature

Swirling

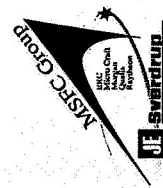
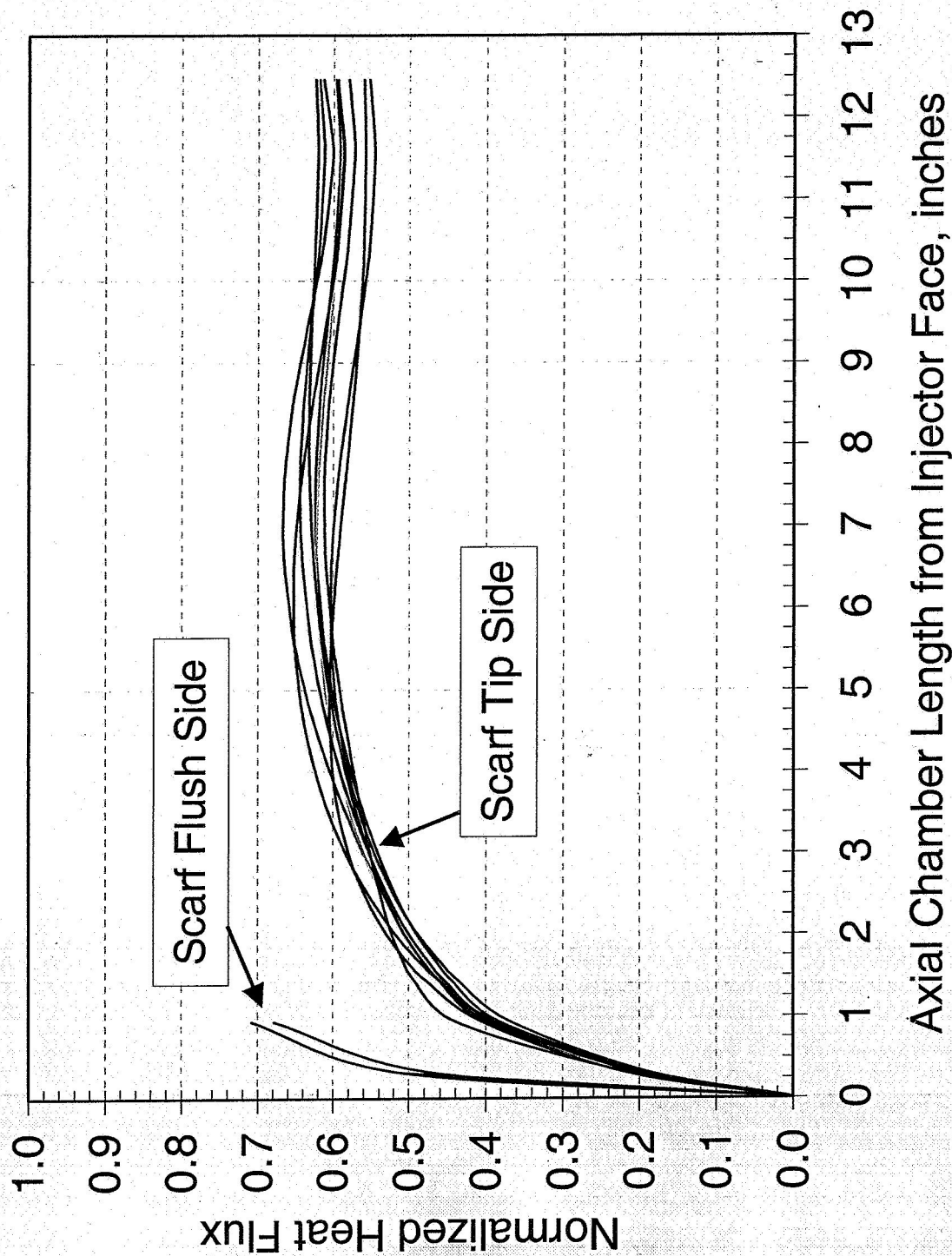
Oxidizer





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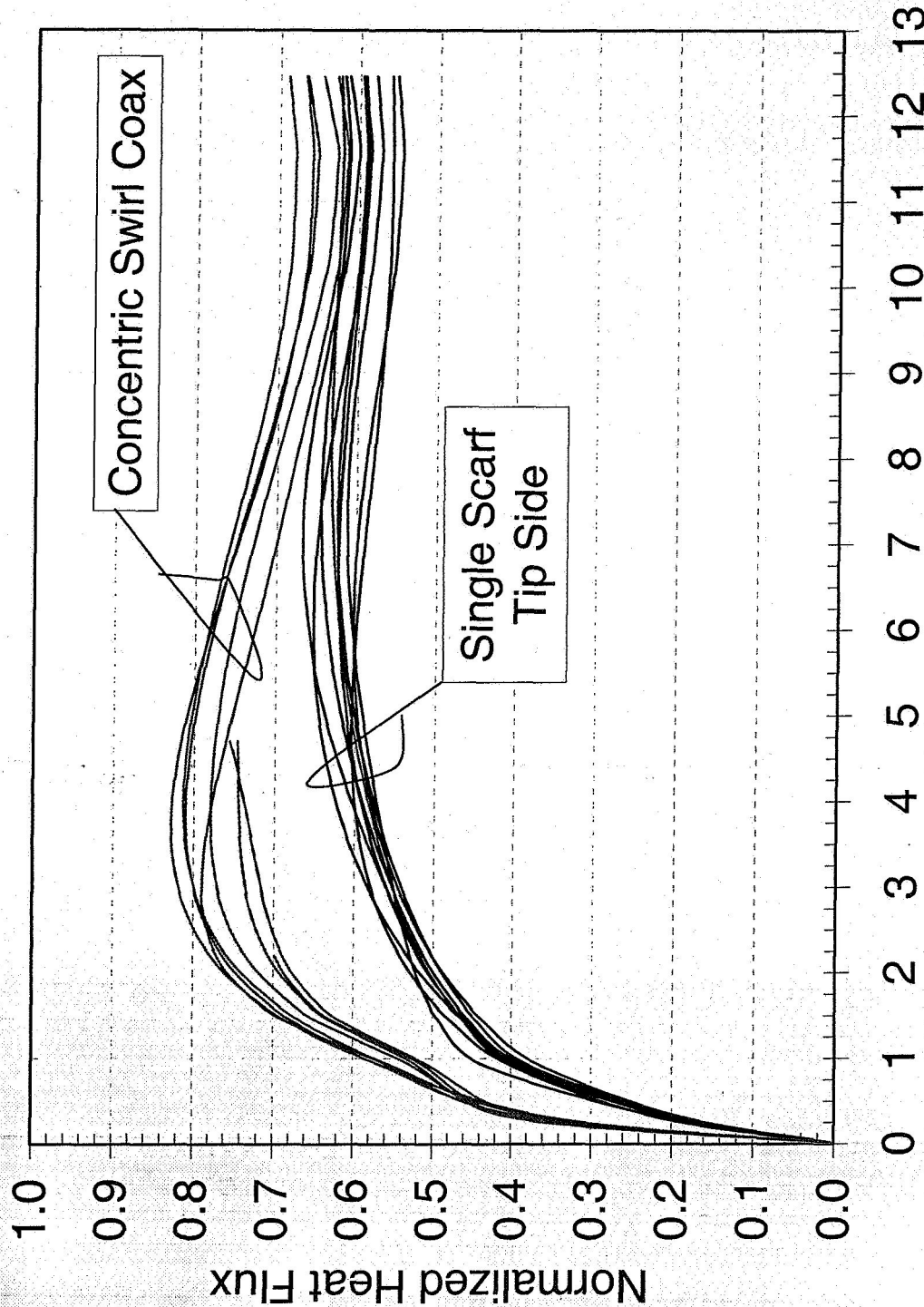
Single Element Scarf Swirl Has Large Circumferential Heat Flux Variation





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“Tip” Side of Scarf Swirl Reduces Heat Flux From Concentric Swirl Coax



Axial Chamber Length from Injector Face, inches



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Acknowledgements

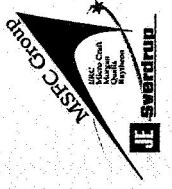
- Funding
 - Meg Tuma, Pete Mazurkivich, Steve Kurtz, Rick Ryan, and Terri Tramel of the NASA MSFC Space Transportation Program and Projects Office
- Testing
 - Sibtosch Pal, Robert Santoro, and William Marshall of The Pennsylvania State University
 - Larry Jones of Medtherm

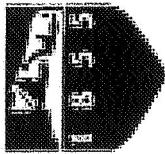




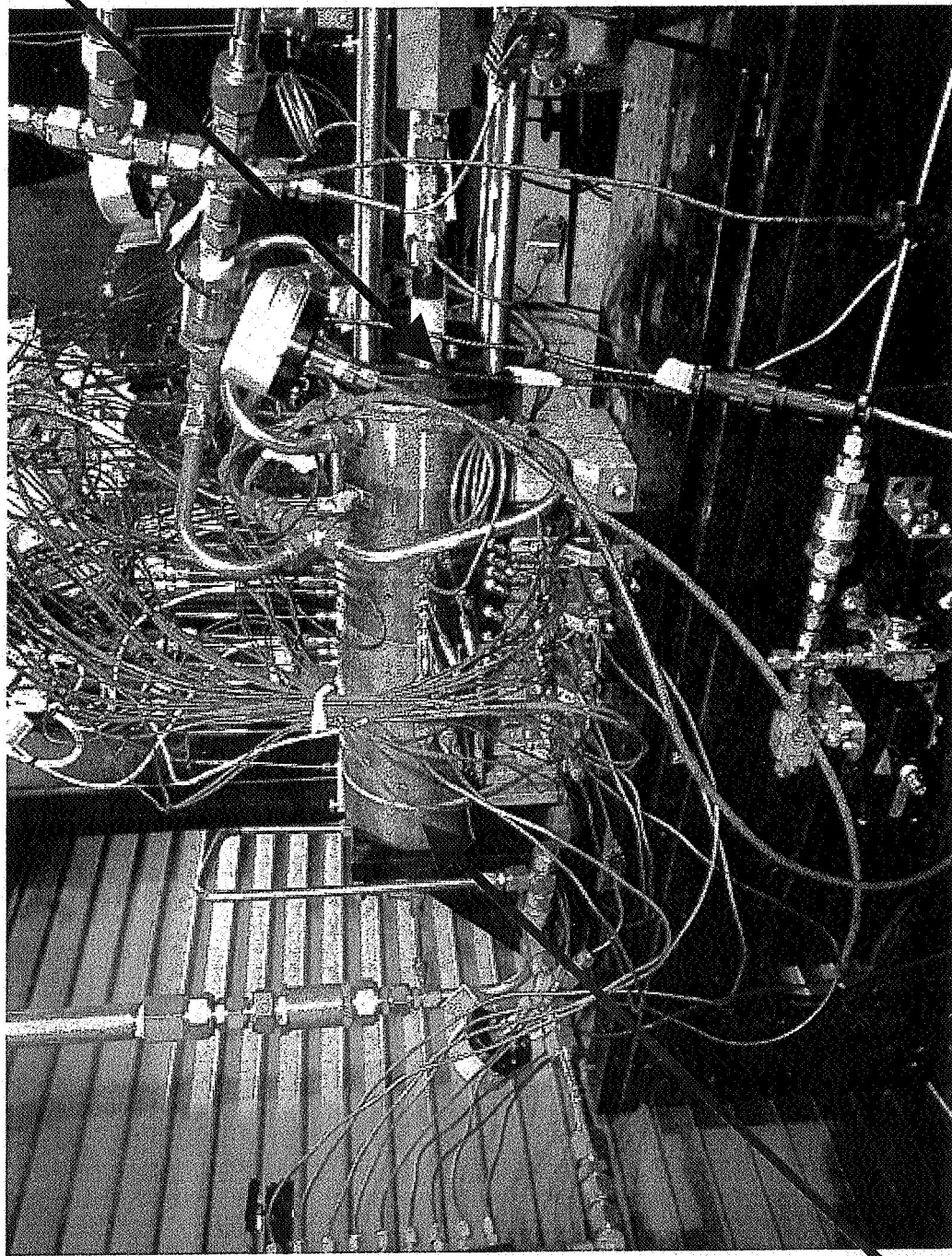
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Extra Slides





Compatibility/Heat Transfer Test Rig at The Pennsylvania State University



Injector

Nozzle



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Combustion CFD Used for Pre-test Experimental Design & Post-Test Code Validation

- Role of CFD in CDIT
 - Pre-test -
 - Guide the experimental design
 - Evaluate scaling relationships
 - Examine injector flowfield features
 - Post-test -
 - Perform code validation
 - Evaluate experimental data quality
- CFD Codes
 - FDNS (Finite Difference Navier Stokes)
 - Used on all calculations to date
 - Benefits - real fluids model, chemistry, previous use for reacting flows
 - Disadvantages - limited to structured grids, inefficient in parallel mode
 - Loci-STREAM
 - To be used pending release
 - Benefits - generalized grids, scales well, Loci-framework
 - Disadvantages - applicable release not available until Fall 2005

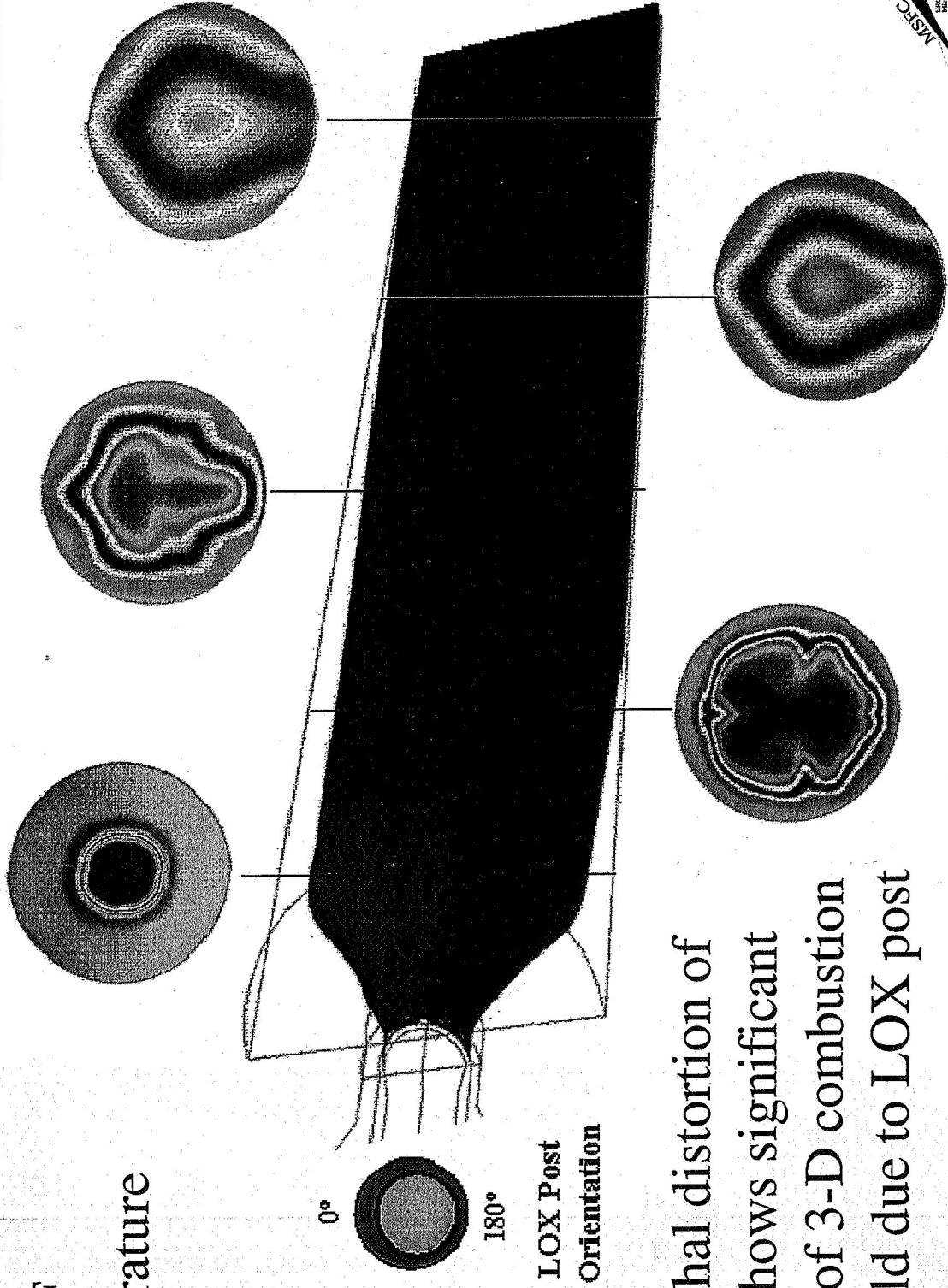


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3-D Combustion CFD of Offset Shear Coax Single Element

- 5000 °F

Iso-temperature surface



- Azimuthal distortion of flame shows significant effects of 3-D combustion flowfield due to LOX post offset



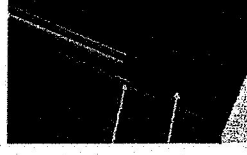
Heat Flux Data Reduction Methodology

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Flat Plate



Cylinder



**Steady State
Heat Conduction**

$$\dot{q} = \frac{k\Delta T}{\Delta x}$$



**Steady State
Heat Conduction**

$$\dot{q} = \frac{k(T_{i2} - T_{o2})}{R_i \ln(R_o/R_i)}$$

**Steady State Heat Conduction
Including Capacitance**

$$\dot{q} = \frac{k(T_{i2} - T_{o2})}{R_i \ln(R_o/R_i)} + \frac{\rho c_p \Delta T (R_o^2 - R_i^2) \ln(R_o/(R_i + \Delta r))}{2 R_i \Delta t \ln(R_o/R_i)}$$

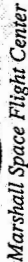
where ΔT is:

$$\Delta T = \{(T_{i2} - T_{i1}) - (T_{o2} - T_{o1})\} \left[\frac{1}{4 \ln(R_i/R_o)} - \frac{R_i^2}{2 (R_i^2 - R_o^2)} \right] + (T_{o2} - T_{o1}) / 2$$

subscripts:

i => inside diameter; o => outside diameter;
1 => time #1; 2 => time #2





- During initial portion of test firing, instrument body is not conducting
- After some time, body becomes well attached to chamber wall
- Near end of test firing, body begins to conduct heat to chamber

