

Active Flow Control Stator with Coanda Surface

**DESIGN OF A LOW SOLIDITY FLOW-CONTROL STATOR
WITH COANDA SURFACE
IN A HIGH SPEED COMPRESSOR**

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AFC Stator with Coanda Surface

0 Introduction

1 Test Facility

2 Concept

3 Aerodynamic Design

4 Mechanical Design

5 Experimental
Results

6 Conclusions



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Introduction:

- Active Flow Control increases the permissible aerodynamic loading
- Curved surface near the trailing edge (“Coanda surface”)
 - increases turning → higher pressure ratio
 - controls boundary layer separation → increased surge margin

Objective:

Reduce the number of vanes or compressor stages.

Constraints:

1. In a real compressor, the vane must still function entirely without blowing.
2. Maintain the flow exit angle of the reference stator despite the resulting increase in stator loading.



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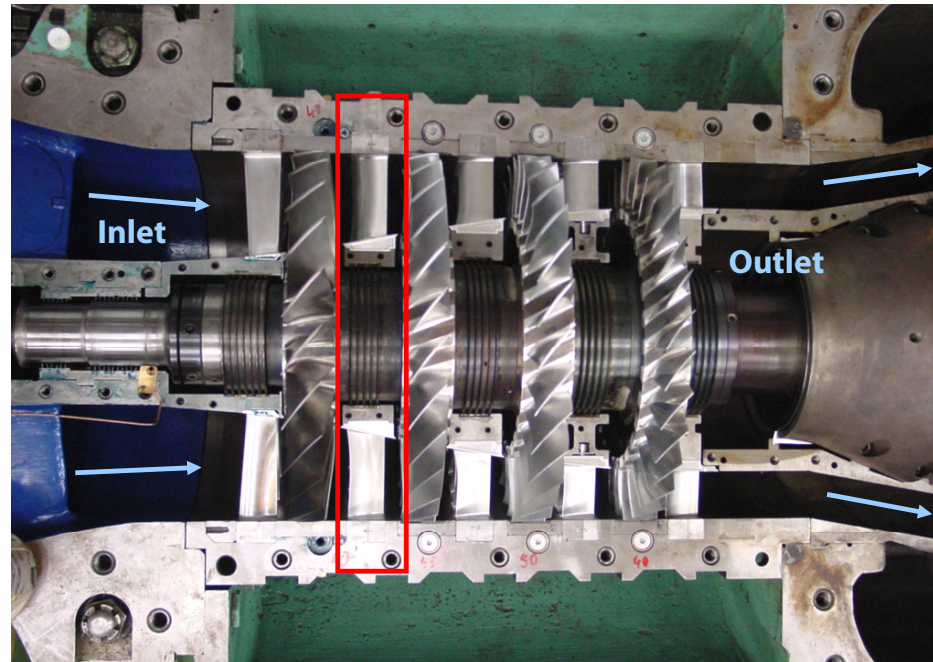
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Compressor test rig at TFD Performance Data



Design speed	17100 rpm
Mass flow	7.81 kg/s
Total pressure ratio	2.75
Isentropic efficiency	90.5%

Power	950 kW
Flow coefficient 1st stage	0.71
Loading 1st stage	0.44
Reynolds number (stator 1)	4×10^5



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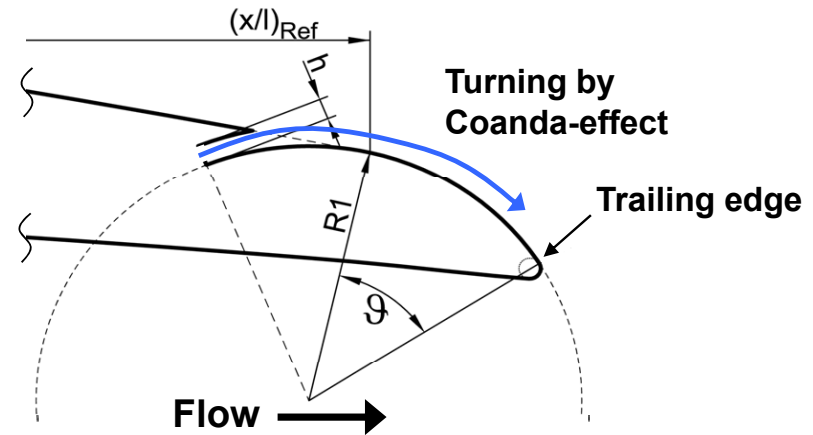
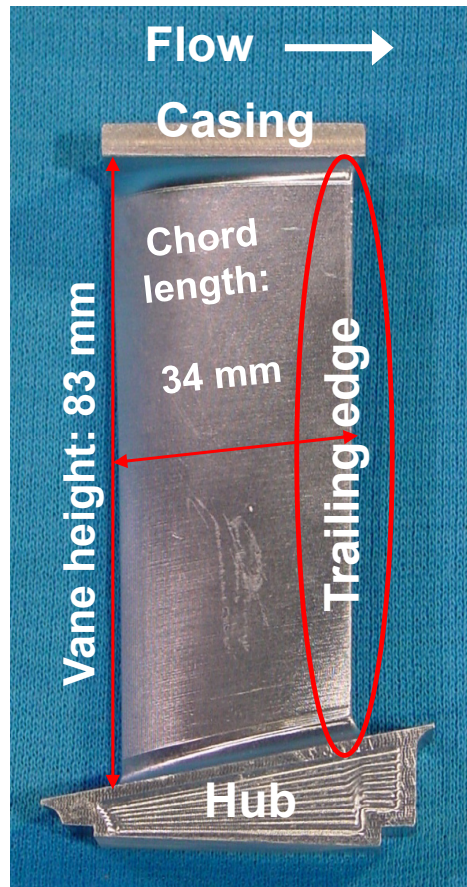
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Concept and Design of the Coanda Stator

Definition of the Coanda Surface at the Trailing Edge



- Chord length in mid-span, Stator 1: $l = 34 \text{ mm}$
- Thickness to chord ratio: 8%
- Slot height: $h = 0.2 \text{ mm}$
- Slot height/Coanda-radius: $h/R1 = 2\%$
- Coanda-radius: $R1 = 10 \text{ mm}$



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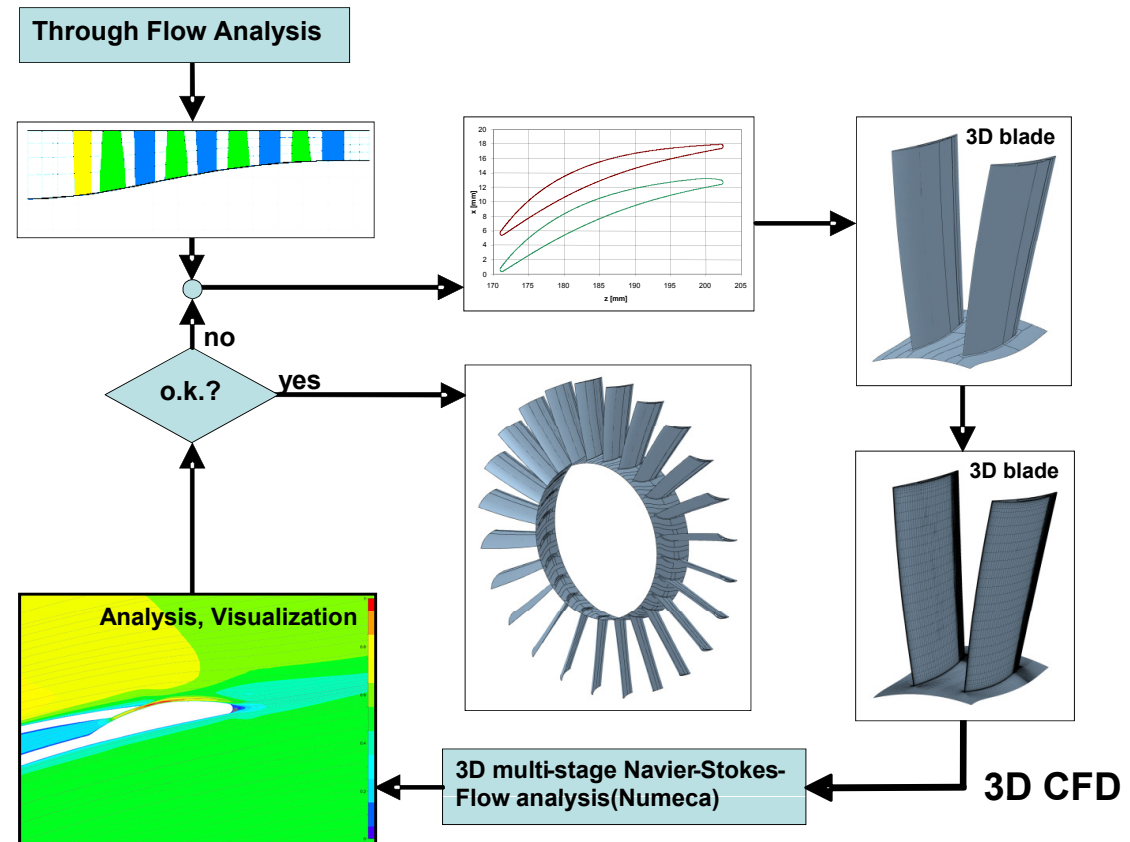
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Concept and Design of the Coanda Stator

Basic Design Loop





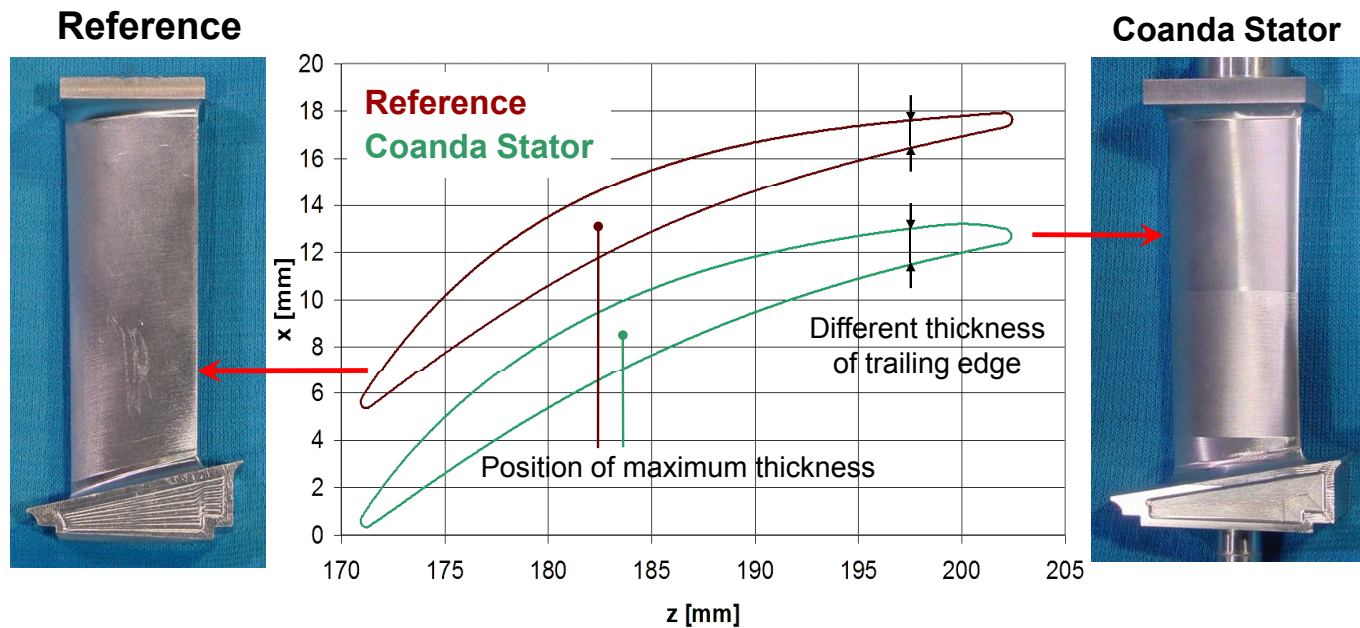
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Concept and Design of the Coanda Stator

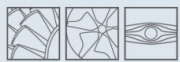
Profile Section in Mid-Span: Reference and Coanda Stator



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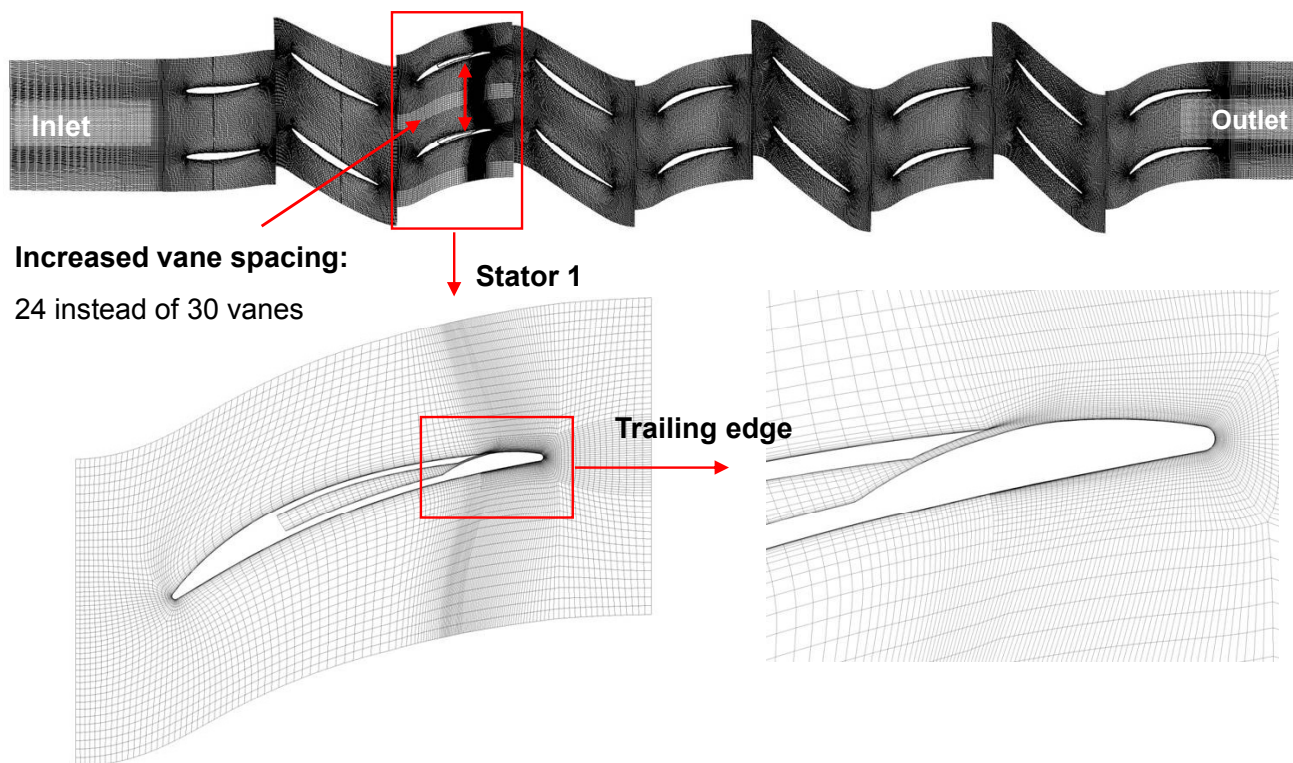


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3D CFD Simulations

Mesh for 3D CFD Simulations: Plenum + Slot





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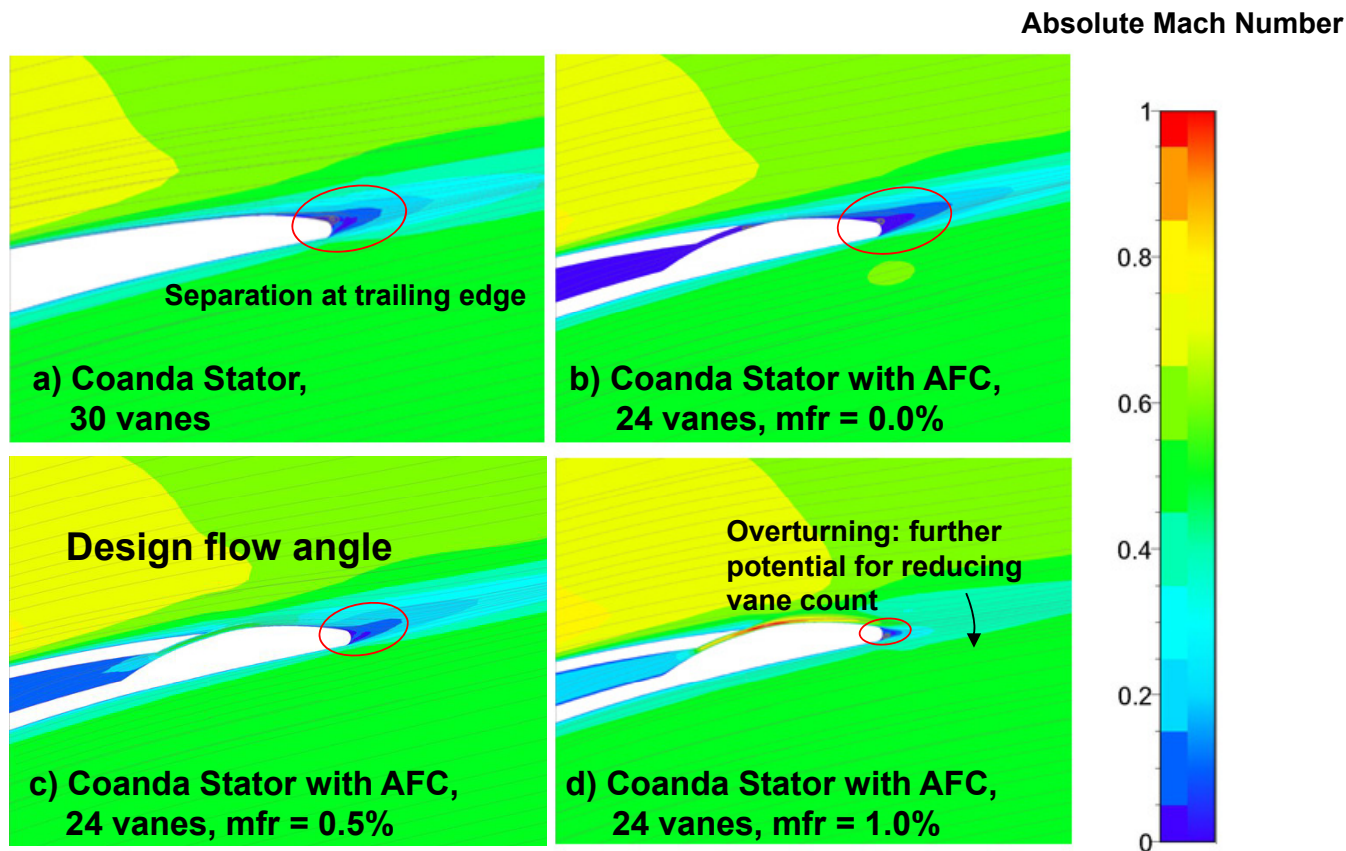
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3D CFD Simulations

Mach Number Distribution around Trailing Edge at Mid-Span



Blowing rate: mass flow ratio (mfr) = jet flow / main flow



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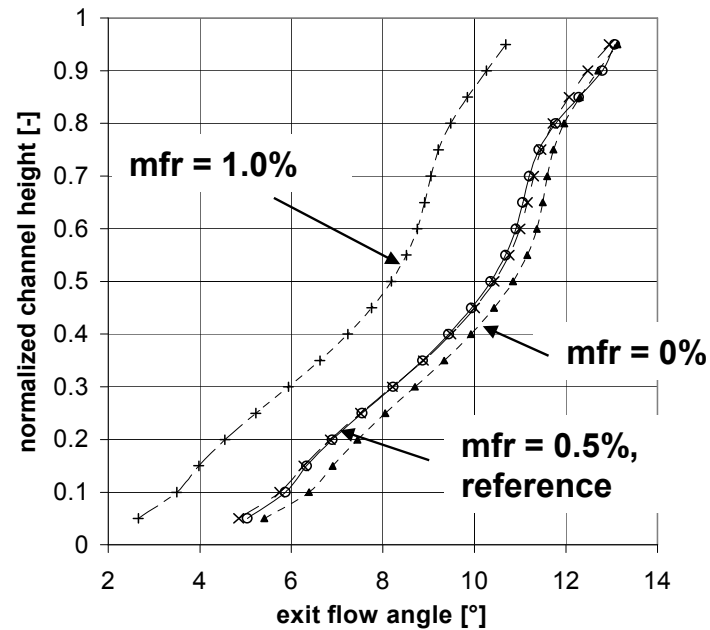
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3D CFD Simulations

Radial Distribution of the Exit Flow Angle



→ Reduction of Compressor Stages



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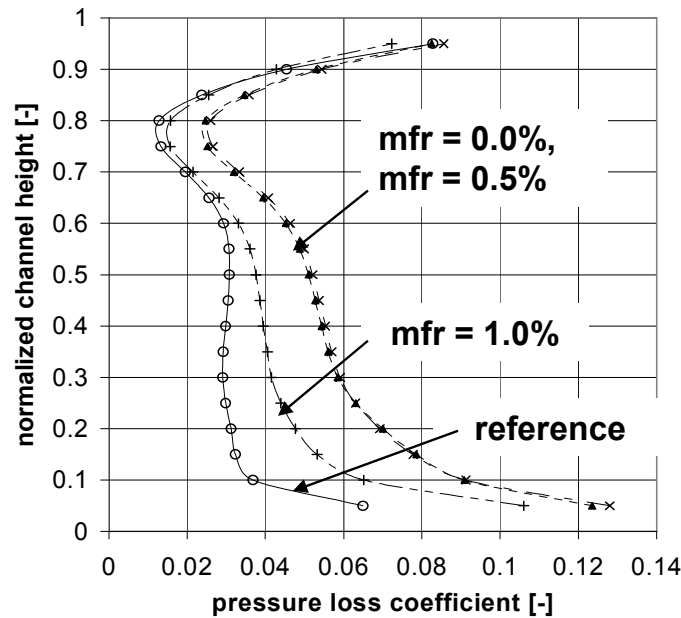
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3D CFD Simulations

Radial Distribution of Total Pressure Loss Coefficient



$$\omega = \frac{P_{tot,1} - P_{tot,2}}{P_{tot,1} - P_{stat,1}}$$

Note: momentum of injected flow not accounted for

Reduction of Losses → Higher Efficiency



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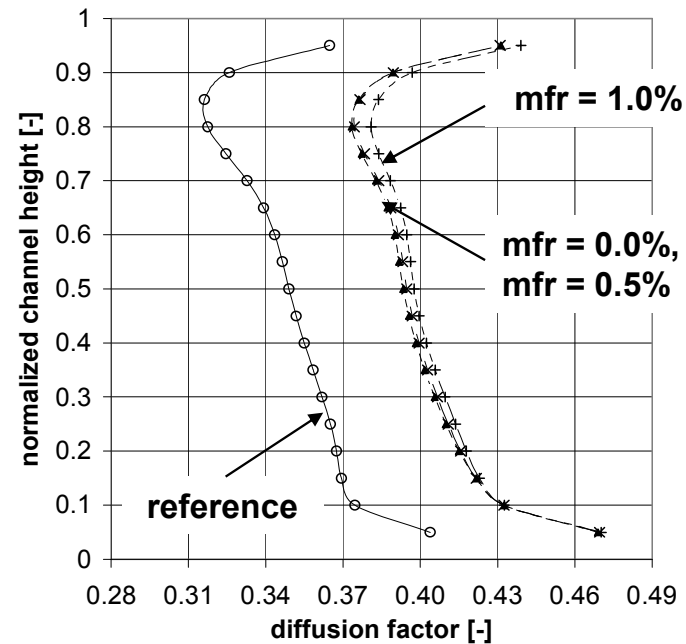
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3D CFD Simulations

Radial Distribution of the Diffusion Factor



$$Df = 1 - \frac{w_2}{w_1} + \frac{v_{\theta 2} - v_{\theta 1}}{2\sigma w_1}$$

**Increased Permissible Aerodynamic Loading
by 13% at the Design Point
→ Better Part Load Performance?**



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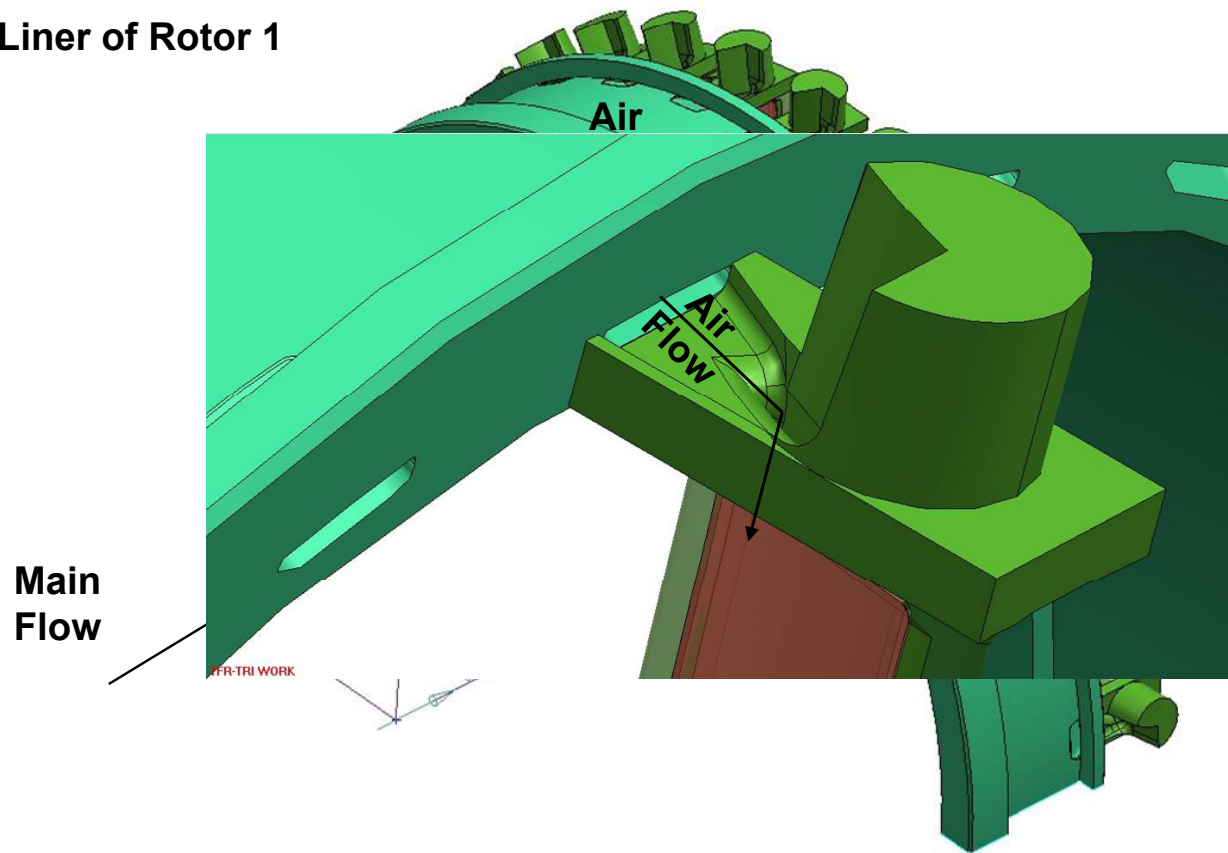
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Design, Manufacture and Integration in Compressor

Air Supply for Flow Control Stator

Liner of Rotor 1



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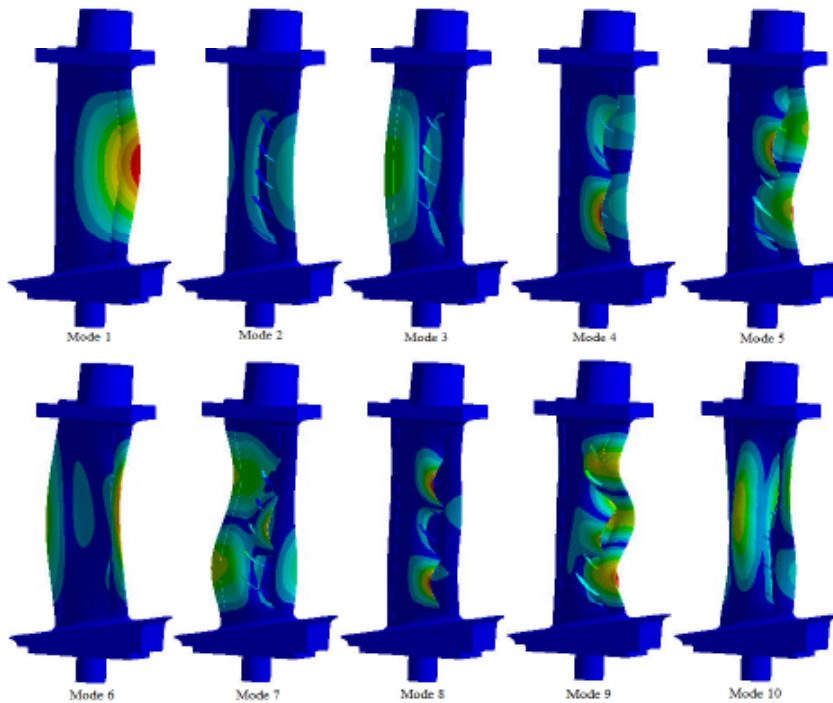
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Design, Manufacture and Integration in Compressor

FEM-Modal Analysis of Coanda Stator



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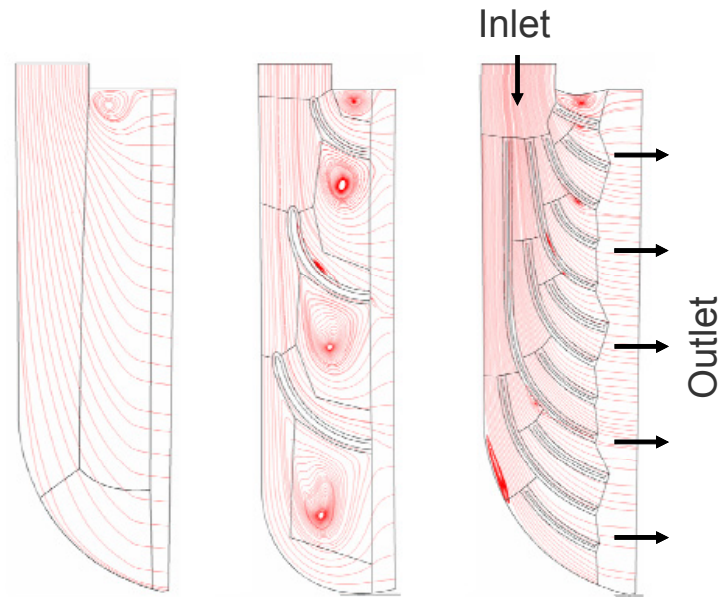
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Design, Manufacture and Integration in Compressor

Iterative Design of Plenum in Coanda Stator



Aerodynamics: loss-minimized flow in plenum of Coanda stator



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Design, Manufacture and Integration in Compressor

Coanda Stator



- Two parts: vane body and cover
- Laser welded: low warpage; durable and leak-proof
- Seamless surface: finish after welding



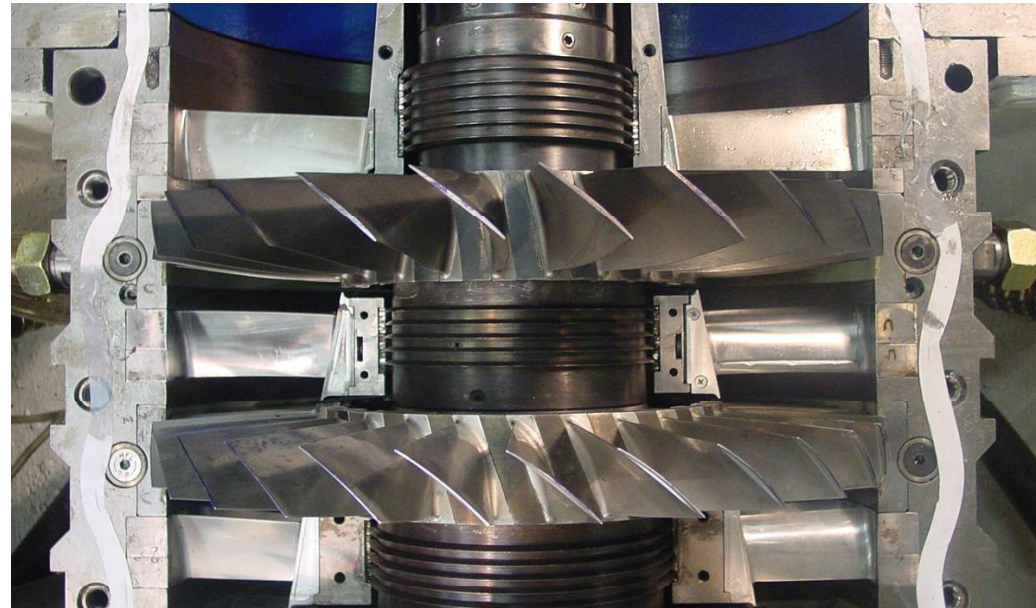
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Design, Manufacture and Integration in Compressor

Coanda Stator in First Stage of Compressor



No change of geometry except for Coanda Stator



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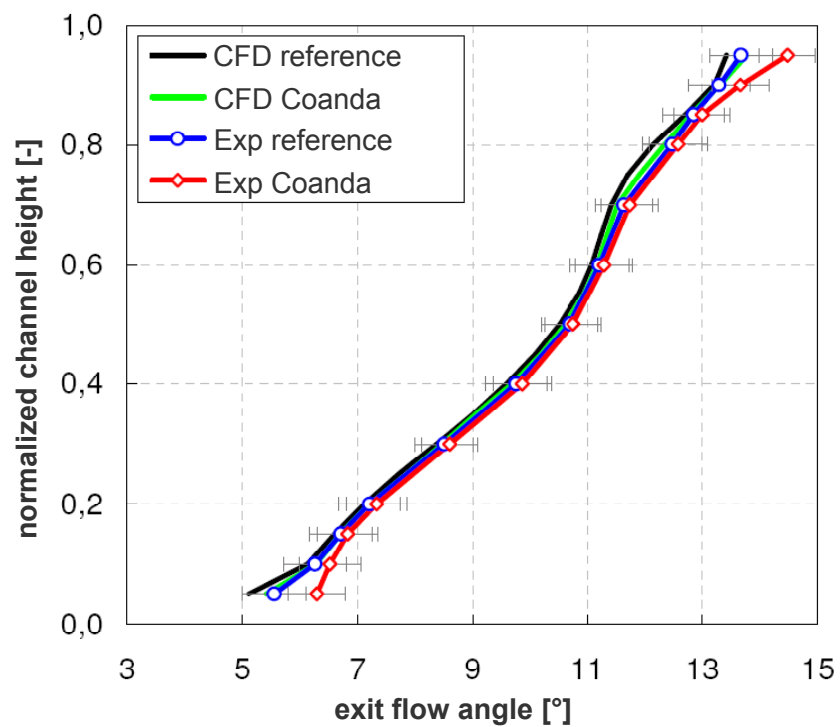
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Radial Distribution of the Exit Flow Angle





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Conclusions

- **Turning of the Coanda-surface augmented by blowing**
 - **Higher stage pressure ratio**

- **0.5% of Compressor Inlet Mass Flow are sufficient to reduce vane count by 20%**
 - **Reduced number of stages, reduced weight, lower investment cost possible**

- **Aerodynamic performance confirmed for aerodynamic design point**
 - **Increased Permissible Aerodynamic Loading by 13%**