

FACE SHEET/CORE DISBOND GROWTH IN HONEYCOMB SANDWICH PANELS SUBJECTED TO GROUND-AIR-GROUND PRESSURIZATION AND IN-PLANE LOADING

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The Aerospace Corporation, El Segundo, CA, June 25, 2015

Work was funded by NASA Langley Research Center under contract number NNL09AA00A

OVERVIEW



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- Detailed Problem Description
- Analysis Methodology
 - Fracture Mechanics Approach
 - Finite Element Modelling
- Initial Model Verification and Validation
- Analysis Results
 - Analysis of a Flat Panel Under Internal Pressure, In-Plane and Combined Loading
 - Analysis of a Curved Panel
- Summary
- Concluding Remarks

BACKGROUND



- Problem
 - In-service component failures associated with disbonding in unvented honeycomb core sandwich
 - Degradation due to disbonding affects operational safety
 - Failures may discourage use of composites in 'future' vehicles
 - Methods for assessing propensity of sandwich structures to disbonding not fully matured, accepted and documented
 - Methods development is currently being discussed within the Disbond/ Delamination Task Group in CMH-17









OBJECTIVE



- Identify, describe and address the phenomenon associated with face sheet/core disbonding
- Increase the knowledge on the subject and the awareness of consequences
- Develop a methodology to assess face sheet/core disbonding in honeycomb sandwich components similar to delamination in composite laminates
 - Develop standard test methods for characterizing face sheet/core disbonding in sandwich components
 - Develop a fracture mechanics based methodology to assess face sheet/ core disbonding in sandwich components
 - Develop models and analysis tools for face sheet/core disbonding in sandwich components subjected to ground-air-ground cycles and/or inplane loading
 - Evaluate the developed test methods and analysis tools using honeycomb sandwich panel tests

DETAILED PROBLEM DESCRIPTION

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- Pressure difference between the in- and outside of unvented sandwich structures
 - Caused by alternating ambient pressure and temperature
 - Results in significant deformations and core volume increase
 - Volume increase results in pressure decrease based on the ideal gas law

pV = nRT

- Initial disbonds between face sheets and core
 - increase the peeling effect and
 - decrease the structural reliability significantly
- For an accurate structural analysis, a coupled pressure-deformation problem needs to be solved

 Initial configuration at ground elevation



 Deformed configuration at cruising altitude



ANALYSIS METHODOLOGY Fracture Mechanics Approach

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- Two steps are required to identify, describe and address face sheet/ core disbonding
 - Test standard development in ASTM committee D30 (WK 47682)
 - Characterize the properties of the face sheet/core interface^[14]
 - \circ Measure fracture toughness G_c
 - Analysis Development
 - Compute the energy release rate along the disbond front
 - Use the Virtual Crack Closure Technique (VCCT) based on the results obtained from a finite element analysis
- Propagation is predicted to occur once the computed value exceeds the measured fracture toughness

[14] reference to publication cited in conference proceedings





ANALYSIS METHODOLOGY Finite Element Modelling – 1/4

- A quarter section of a flat panel was modelled
 - Circular disbond radius: 152.4 mm (6")
 - Square section modelled: 304.8 mm (12")
 - Abaqus/Standard[®] was used (C3D20)
 - Boundary conditions applied at symmetry planes
 - Surface contact used between top face sheet and core in the disbonded section
- Sandwich properties based on previous results
 - Thin face sheet: 0.772 mm (0.03")
 - CYCOM 5320PW plain weave fabric
 - [45/0/90/-45] quasi-isotropic layup
 - Thick core: 76.5 mm (3.0")
 - Hexcel HRH-10® honeycomb
 - NOMEX[®] paper with 48 kg/m³ (3.0 lb/ft³) density and 3.175 mm (1/8") cell size
 - Modelled as an orthotropic, homogeneous continuum





ANALYSIS METHODOLOGY Finite Element Modelling – 2/4



- Pressure deformation coupling was simulated using fluid filled cavities
 - Abaqus/Standard[®] feature enabled the definition of fluid-filled cavities enclosed by structural elements
 - The ideal gas law is solved within each increment until equilibrium is found
 - The volume of the fluid cavities was assumed to be equal to that of the entire sandwich core
 - Two separate cavities were defined
 - One cavity was used to simulate the intact part
 - The other cavity included only the disbonded section
 - The disbonded cavity extended by one cell size, 3.175 mm (1/8"), ahead of the disbond front





ANALYSIS METHODOLOGY Finite Element Modelling – 3/4



- Model of a flat panel with in-plane loading
 - Study the effect of in-plane service load on a flat control surface
 - In-plane displacement applied to the model to simulate a 0.2% (2000 με) strain condition during a flight maneuver
 - A compressive strain condition was chosen since it was believed that it would aggravate the condition
- Model of a curved panel
 - Honeycomb sandwich constructions may be used for cylindrical fuselage structures
 - A 3 m radius (wide body airliner) was chosen for this study



ANALYSIS METHODOLOGY Finite Element Modelling – 4/4

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- Internal pressurization of the disbond
 - Commercial jetliner ascent scenario was considered from 0 to 12192 m (0 to 40000 ft).
 - The pressure and temperature values were taken from the International Standard Atmosphere ISO 2533
 - The temperature in the core was defined to be equal to the ambient temperature
 - Pressure and volume inside the cavities were calculated during the analysis
- Additional load conditions
 - 0.2% (2000 $\mu\epsilon$) strain condition only
 - Combination of GAG and 0.2%
 (2000 με) strain

Decrease of temperature and pressure with increasing altitude



Flat panel under internal pressure loading – 1/3

- Initial study^[6]
 - Variation of
 - Face sheet thickness, number of plies
 - o Disbond radius
 - Core density: 29 kg/m³, 48 kg/m³, 80 kg/m³ (1.8 lb/ft³, 3.0 lb/ft³, 5.0 lb/ft³)
 - Core thickness: 12.5 mm,
 25.4 mm, 50.8 mm, 76.5 mm
 (0.5" 3.0")
 - Results
 - $\circ~$ Variation of core density does not have a significant effect on computed $G_{\rm T}$
 - $\circ~$ Large disbond radius and thin face sheets result in maximum G_{T}

Current study

 Dimensions based on results from initial study Averaged G_T along crack front

3.275 mm (1/8") cell size, 48 kg/m³ (3.0 lb/ft³) core density





Flat panel under internal pressure loading – 2/3

Conditions

- 12,192 m altitude (40,000 ft)
 - External pressure p=0.0188 MPa
 - External temperature T= 216.65 K
- Verification for using a FE model of a quarter section of the panel
 - Analysis using a full model of the panel with circular disbond
 - Analysis using a model of a quarter panel with boundary conditions
 - Excellent agreement of computed G_T along the front for the currently used quasi-isotropic layup
 - Deviation, however, for other layups that violate the symmetry conditions of the model

• Distribution of energy release rate along the disbond front



Flat panel under internal pressure loading – 3/3

- **Conditions** •
 - 12,192 m altitude (40,000 ft)
 - External pressure p=0.0188 MPa
 - External temperature T= 216.65 K 0
- Result •
 - Max G_T observed at φ=45°

Conditions •

- 0 m 12,192 m altitude
- Sea level to cruising altitude

Results for max G_{τ} at ϕ =45° •

 G_{T} increases monotonically with increasing altitude



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Flat panel under in-plane and combined loading

Conditions

- 12,192 m altitude (40,000 ft)
 - External pressure p=0.0188 MPa
 - External temperature T= 216.65 K
- 0.2% (2000 με) applied in-plane strain to simulate service loads on a flat control surface
- Combined internal pressure + 0.2%
 (2000 με) in-plane strain

Results

- Out of plane deformation of the disbonded section changes
- Leads to a change in the G_T distribution
- In-plane strain aggravates the condition
- Due to non-linearity superposition of the results is not possible

• Distribution of energy release rate along the disbond front



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Analysis of a curved panel

Conditions

- 12,192 m altitude (40,000 ft)
 - External pressure p=0.0188 MPa
 - External temperature T= 216.65 K
- Flat panel
- Curved panel with 3 m radius

Results

- Symmetry of the G_T distribution is lost for the curved panel
- Locally and on average the computed G_T is higher than the result obtained from the flat panel
- Result is unexpected
- In-plane strain may further aggravate the condition
- Additional analyses with different radii and more refined mesh should be preformed before a definite statement is made

• Distribution of energy release rate along the disbond front



SUMMARY



- A sandwich panel containing a circular disbond at the face sheet/core interface was studied.
- A fracture mechanics approach was used.
- The pressure-deformation coupling was a focus of the analysis.
- Special fluid-filled cavities were used to model the entrapped air.
- Sandwich panels with large disbonds, thin face sheets, and thick cores are most critical.
- Computed averaged energy release rate values increased almost linearly with increasing altitude.
- The presence of the in-plane compressive strain aggravated the condition along the crack front.
- Due to the non-linearity of the problem, the results for combined load cases cannot simply be obtained by superposition of the individual load cases.
- For a curved panel with 3 m radius, the computed energy release rate values were higher than the values computed for a flat panel.

CONCLUDING REMARKS



- Overall, the finite element analysis with fluid cavities appears to perform well and is capable of capturing the pressure-deformation coupling in the disbonded section of the panel.
- Based on the current preliminary results, however, it is recommended that additional validation studies be performed to compare.
 - The computed local deformation field of the disbonded face sheet with far field measurements
 - The computed pressure inside the cavity with measured values.
- Additionally, analyses of curved panels with different radii should be performed before a definite statement about the effect of panel curvature on the crack tip loading is made.
- Methods development will continue within the Disbond/Delamination Task Group in CMH-17

ACKNOWLEDGEMENTS

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The analyses were performed at the Durability, Damage Tolerance and Reliability Branch at NASA Langley Research Center, Hampton, Virginia, USA while Zhi Chen was a participant in the Langley Aerospace Research Student Scholars (LARSS) program. Ronald Krueger (NIA) was supported under contract NNL09AA00A and Martin Rinker was a visiting scientist at the National Institute of Aerospace (NIA).



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BACKUP SLIDES



INITIAL MODEL VERIFICATION AND VALIDATION – 1/2

1200



- X-33 cryogenic fuel tank
 - NASA sandwich disbond investigation^[3]
 - Square delamination
 - Panel pressurized by a compressor
 - Defined load, no pressuredeformation coupling
 - Calculations were performed using surface loads
 - Current analysis approach^[6]
 - Same dimensions as NASA publication
 - Pressure application with Abaqus fluid elements
 - VCCT calculation using postprocessing routine

- Result comparison
 - $\circ \quad \mbox{Good correlation between } G_{\rm T} \mbox{ values } \\ \mbox{ calculated using different models } \\$



INITIAL MODEL VERIFICATION AND VALIDATION – 2/2

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Sandwich panel with disbond

- Airbus test in vacuum chamber^[4]
 - Panel with 350 mm disbond
 - Pressure-deformation coupling needs to be considered
 - Pressure in disbonded core section was measured during test
 - FE analysis was performed calculating pressure-deformation coupling iteratively



- Current analysis approach
 - Same dimensions as Airbus panel
 - Pressure pressure-deformation
 coupling solved with Abaqus fluid
 elements
- Result comparison
 - Pressure-deformation coupling is correctly solved via Abaqus Fluid Cavity Simulation
 - Pressure in core:
 - Airbus test: 0.0582 MPa
 - \circ Airbus analysis: 0.0577 MPa
 - o Current analysis: 0.0571 Mpa
- Additional validation studies should be performed to compare test results and analysis
 - Compare deformation field
 - Compare pressure inside the cavity