

Budget of Turbulent Kinetic Energy in a Shock Wave/Boundary-layer Interaction

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Implicit large-eddy simulation (ILES) of a shock wave/boundary-layer interaction (SBLI) was performed. Quantities present in the exact equation of the turbulent kinetic energy transport were accumulated and used to calculate terms like production, dissipation, molecular diffusion, and turbulent transport. The present results for a turbulent boundary layer were validated by comparison with direct numerical simulation data. It was found that a longer development domain was necessary for the boundary layer to reach an equilibrium state and a finer mesh resolution would improve the predictions. In spite of these findings, trends of the present budget match closely with that of the direct numerical simulation. Budgets for the SBLI region are presented at key axial stations. These budgets showed interesting dynamics as the incoming boundary layer transforms and the terms of the turbulent kinetic energy budget change behavior within the interaction region.

Nomenclature

α	collision frequency
δ_{99}	boundary-layer thickness
δ_{ij}	Kronecker delta
ϵ	dissipation rate
γ	specific heat ratio
κ	thermal conductivity constant
\mathbf{E}	electric field vector
$\mathbf{F}, \mathbf{G}, \mathbf{H}$	flux vectors
\mathbf{I}	identity tensor
\mathbf{Q}	heat conduction vector
\mathbf{S}	source vector
\mathbf{U}	solution vector
\mathbf{V}	velocity vector
D_ν	molecular diffusion
D_p	pressure diffusion
\mathcal{M}	mass flux
\mathcal{P}	production
\mathcal{T}	turbulent transport
μ	molecular viscosity
∇	gradient operator
ω	specific turbulent dissipation rate
Π	pressure dilatation

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ρ	density
ρ_c	counterflow force model charge density
$\tau_{ij}, \boldsymbol{\tau}$	Reynolds-stress tensor
θ	boundary-layer momentum thickness
θ_f	frequency of applied voltage
ϑ	counterflow force effective duty cycle
ζ	second viscosity
C_p	specific heat at constant pressure
D_c	counterflow force model coefficient—ratio of electrical to inertial forces
e_0	total energy
e_c	electronic charge
k	turbulent kinetic energy
L	length scale
M	Mach number
p	pressure
Pr	Prandtl number
Re	Reynolds number
S_{ij}	mean strain-rate tensor
s_{ij}	instantaneous strain-rate tensor
T	temperature
t	time
t_{ij}	viscous stress tensor
u, v, w	non-dimensional velocity in x, y, z directions, respectively
V	velocity magnitude
x, y, z	non-dimensional coordinates

Subscripts

0	total condition
i, j, k	index notation, equal to 1, 2, 3
inv	inviscid
ref, ∞	reference value same as freestream value
vis	viscous
w	wall

Conventions

—	Reynolds averaged
~	Favre averaged

Superscripts

'	Reynolds decomposition
"	Favre decomposition
*	non-dimensional value
+	non-dimensionalized by inner scales
tr	transpose

I. Introduction

Shock wave/boundary-layer interactions (SBLIs) have a rich history of research and advancement over the course of the last 60 years in the areas of experimentation, modeling, and simulation.^{1–3} SBLIs are ubiquitous in transonic, supersonic, and hypersonic flight, and thus relevant to the majority of commercial, military, and space vehicles of the past, present, and future. They can be found on the external body, like the vehicle nose, wings, and/or tail, and internal to the propulsion flowpath, which includes mixed compression inlets, diffusers, and isolators. Aerothermal loads created by SBLIs can compromise structural integrity, cause component failure, and result in loss of control. Adverse pressure gradients in the propulsion flowpath can cause flow distortion and lead to loss in engine efficiency. Because of these characteristics, study of SBLIs is one of the most active areas of research in high speed flows. But despite the collection of work pursued in

industry, academia, and government laboratories, a complete understanding of SBLIs remains illusive, and hence their prediction via modeling and simulation is difficult.

The challenges in modeling and simulation are multifold—SBLIs are unsteady in nature, the resulting separation is highly three-dimensional (3D), and corners often separate (e.g., propulsion flowpaths are often rectangular). A combination of these lead to a locally inhomogeneous flowfield. A prominent debate in SBLI research involves the source of low-frequency unsteadiness of the reflected (or separation) shock, which is typically one or two orders of magnitude lower than the frequencies within the incoming turbulent boundary layer.^{1,4,5} Although much experimental and computational work has been done to study the source of this unsteadiness no clear consensus has been reached. Questions remain: Is the source of low-frequency unsteadiness located upstream⁶ or downstream^{7,8} of the interaction? Or is it a coupled system influenced by sources present upstream, downstream, and within the separation bubble?⁴ An elementary understanding of separation and corner influence exists, however modeling improvements from such understanding has yet to be realized. It is this uncertainty in the knowledge of SBLI physics which make them extremely difficult to model and simulate. An example of a canonical two-dimensional (2D) SBLI flowfield is shown in figure 1. Fluctuations in the incoming turbulent boundary layer can be classified as upstream sources, while the unsteadiness of the near-wall/viscous portion of the reflected shock and separation bubble can be classified as downstream sources.

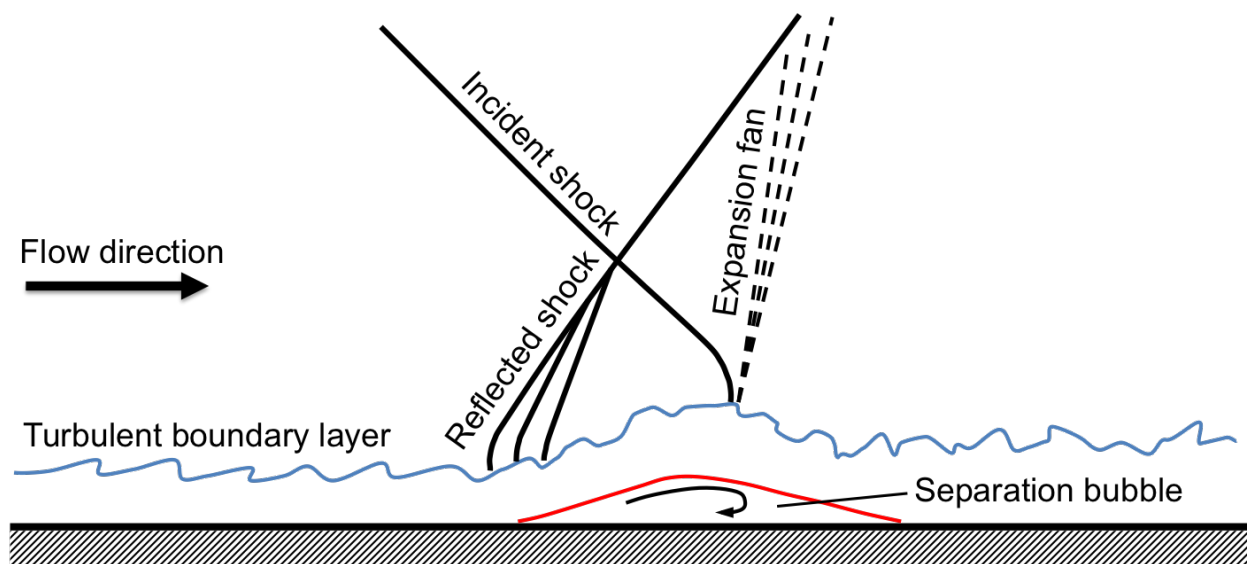


Figure 1. Two-dimensional anatomy of a shock wave/boundary-layer interaction

In most practical applications, Reynolds-averaged Navier-Stokes (RANS) computational fluid dynamics (CFD) solvers coupled with turbulence models are used to calculate flowfields where SBLIs are present. Although as the name suggests, Reynolds averaging or taking a mean of the Navier-Stokes (NS) equations renders them futile, by definition, if the interest lies within exploring the low-frequency unsteadiness driving the reflected shock. Thus, RANS CFD is impervious to unsteadiness by its fundamental design, which could be a factor affecting SBLI predictions. Nonetheless, RANS CFD has been widely used in industry and academia to obtain SBLI predictions due in part to its ease of use, and also to the challenges presented by scale-resolving methods like hybrid RANS/large-eddy simulations (LES), LES, and direct numerical simulations (DNS) in the form of available computer resources and lengthy simulation times.

Yet another approach would be unsteady RANS (URANS), where a global timestep is used to march the solution forward. If it is the low-frequency unsteadiness of the reflected shock that we wish to study, URANS can possibly capture such phenomenon as the reflected shock frequency is on the order of hundreds of Hertz.^{1,7} The incoming turbulent boundary layer as well as the separation bubble exhibit frequencies in the range of 10-40 kHz.^{1,9} Since such a separation of scale exists between the two, in theory, URANS can capture the low-frequency unsteadiness. However, URANS would still require the use of turbulence models, which account for large variations in predictions.¹⁰ Thus, URANS would not be fundamentally superior to RANS.

A description of scale-resolving approaches is provided by Georgiadis et al.¹¹ in a paper that provides a summary of current practices in LES. In LES, large-scale structures are resolved and a sub-grid scale model is employed to model the scales which cannot be resolved by the mesh. A subset of LES is implicit LES (ILES). Like LES, ILES calculates the large-scale turbulent structures, but it does not explicitly model the smallest scales, instead it uses a high-order low-pass filter to dissipate energy in the high spatial wavenumber range where the turbulent energy spectrum is poorly resolved.^{12,13} So, the use of an explicit sub-grid scale model is completely avoided. Thus, ILES is an attractive approach for this work as it is not as expensive as DNS, but does provide a seamless changeover to DNS as the mesh resolution is refined.

A workshop was organized by the American Institute of Aeronautics and Astronautics (AIAA) with an intent to share, assess, and determine the most promising SBLI prediction methods in 2011. The results obtained by the participants were compiled by DeBonis et al.¹⁰ in which comparisons with experimental data and error metrics were presented. While the majority of solutions were obtained with RANS methods, some were obtained with hybrid RANS/LES, LES, and DNS. It was concluded that *the turbulence model played a significant role in variations among different RANS solutions*. In general, the Menter Shear Stress Transport (SST)¹⁴ and Wilcox models,¹⁵ which are $k - \omega$ based two-equation models, and the one-equation Spalart-Allmaras (SA) model, compare well with each other, however *the error in all solutions increased as the adverse pressure gradient becomes stronger and the size of the separation increased*. It was also found that the scale-resolving methods provided some of the best and the worst solutions, clearly indicating that scale-resolving methods are feasible and accurate but that more development is needed.

Perhaps the most interesting revelation DeBonis et al.¹⁰ presented was the fact that the accuracy of a solution was not consistent for different variables of interest within the same solution, i.e. high prediction accuracy in \bar{u} velocity did not guarantee the same accuracy for \bar{v} velocity. This, combined with the other observations above, shows shortfalls of the current one- and two-equation turbulence models. Common among most turbulence models in use today is the Boussinesq eddy-viscosity approximation, which establishes a linear relation between the Reynolds-stress tensor, τ_{ij} , and the mean strain-rate tensor, S_{ij} . Such a relationship does not exist in shock-separated flows such as SBLI, flows where rapid changes in mean strain-rate occur, and where secondary flows are present—all examples of flows which are inhomogeneous and anisotropic.

Some efforts have been made to incorporate the above effects in the existing turbulence models by modifying the turbulent kinetic energy, k , and dissipation rate, ϵ , equations to account for inhomogeneity and anisotropy. Hamlington and Dahm¹⁶ addressed this by replacing the mean strain-rate tensor in a standard two-equation approach with a new mean strain-rate tensor that accounts for flow history, thus allowing the Reynolds-stress tensor to adjust over a finite lag while retaining the simplicity of a two-equation formulation. Sinha et al.¹⁷ included additional terms representing shock unsteadiness in the k and ϵ equations. A linear analysis was used to model the unsteadiness in the $k - \epsilon$ model by realizing that a positive fluctuation in the streamwise velocity leads to the reflected/separation shock motion downstream, while a negative fluctuation in the streamwise velocity causes the reflected/separation shock to move upstream. The improved turbulence model predicted k over a Mach number range of 1.29-3.0 reasonably well and showed improvement over a realizable $k - \epsilon$ model with a realizability constraint of $0 \leq \widetilde{u''u''} \leq 2k$. Numerous other improvements have been suggested to the one- and two-equation turbulence model formulations, though discussing them is beyond the scope of this paper. Another approach is to completely bypass the linear relationship between the Reynolds-stress tensor and the mean strain-rate tensor in one- and two-equation formulations for nonlinear constitutive relations. In some of the more advanced techniques, a direct prescription of the Reynolds-stress tensor is sought using nonlinear algebraic equation or by using Reynolds-stress transport model. However, among these approaches, none has emerged as a clear winner yet.

It is the authors' belief that a physical understanding of the various terms in the exact equation of turbulent kinetic energy transport and their interactions with each other would shed light on the fundamental mechanisms present in SBLI. This knowledge may be used to improve the current turbulence models and/or propose new models. In the past, such efforts involved DNS studies of the turbulent kinetic energy budget on flat plate boundary layers¹⁸ or channel flows¹⁹ with no direct comparison with SBLI flowfields. In the present work, we take a step back from modeling and use ILES to study the budget of turbulent kinetic energy and other relevant turbulence quantities with an objective of informing future turbulence model development for better predictions of SBLIs.

II. Governing Equations

In this section the turbulent kinetic energy transport equation will be discussed along with the equations pertinent to the ILES formulation and the counterflow force model, which is used to obtain a turbulent boundary layer.

II.A. Turbulent Kinetic Energy

The turbulent kinetic energy is defined as

$$k = \frac{1}{2} \widetilde{u''_i u''_i} \quad (1)$$

The transport of turbulent kinetic energy, a standard part of two-equation turbulence models, is given by equation 2. The first term on the left-hand side represents the unsteady term, while the second term represents the convection—together, the left-hand side is the substantial derivative. The budget terms are on the right-hand side.

$$\frac{\partial(\bar{\rho}k)}{\partial t} + \frac{\partial(\bar{\rho}\tilde{u}_j k)}{\partial x_j} = \mathcal{P} + \mathcal{T} + \mathcal{D}_\nu - \bar{\rho}\epsilon + \mathcal{D}_p + \Pi + \mathcal{M} \quad (2)$$

Each term on the right-hand side of equation 2 is defined as

$$\mathcal{P} = - \widetilde{\bar{\rho} u''_i u''_j} \frac{\partial \tilde{u}_i}{\partial x_j} \quad \text{Production} \quad (3)$$

$$\mathcal{T} = - \left(\overline{\rho u''_j \frac{1}{2} u''_i u''_i} \right)_{,j} \quad \text{Turbulent Transport} \quad (4)$$

$$\mathcal{D}_\nu = \left(\overline{t_{ij} u''_i} \right)_{,j} \quad \text{Molecular Diffusion} \quad (5)$$

$$\bar{\rho}\epsilon = \overline{t_{ij} u''_{i,j}} \quad \text{Dissipation} \quad (6)$$

$$\mathcal{D}_p = - \left(\overline{p' u''_i} \right)_{,i} \quad \text{Pressure Diffusion} \quad (7)$$

$$\Pi = \overline{p' u''_{i,i}} \quad \text{Pressure Dilatation} \quad (8)$$

$$\mathcal{M} = \overline{u''_i} (\bar{t}_{ij,j} - \bar{p}_{,i}) \quad \text{Mass Flux} \quad (9)$$

The **production** term represents the rate of transfer of kinetic energy from the mean flow to the turbulence or alternatively, generation of turbulent kinetic energy, which results from the mean velocity gradients in the flowfield. The propagation of the turbulent kinetic energy by the means of triple correlation of velocity fluctuations is given by the **turbulent transport** term. The **molecular diffusion** term, also referred to as viscous diffusion, is responsible for molecular transport of the turbulent kinetic energy. The conversion of the turbulent kinetic energy to thermal internal energy is represented by the **dissipation** term. The **pressure diffusion** term is a form of transport occurring due to diffusion mechanism as a result of pressure and velocity-gradient interaction. Finally, the compressibility terms, **pressure dilatation** and **mass flux**, appears in the compressible form of the turbulent kinetic energy transport equation.

Here t_{ij} is the viscous stress tensor based on the instantaneous strain-rate tensor s_{ij} . And δ_{ij} is the Kronecker delta.

$$t_{ij} = 2\mu s_{ij} + \zeta \frac{\partial u_k}{\partial x_k} \delta_{ij} \quad (10)$$

In equation 10, ζ is obtained by relating it to μ . Such an assumption is valid for monatomic gases and widely used in computational fluid dynamics.

$$\zeta = -\frac{2}{3}\mu \quad (11)$$

II.B. Compressible Navier-Stokes Equations

The compressible Navier-Stokes equations in non-dimensional form are given by

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{V}) = 0 \quad (12)$$

$$\frac{\partial(\rho \mathbf{V})}{\partial t} + \nabla \cdot (\rho \mathbf{V} \mathbf{V} + p \mathbf{I}) - \frac{1}{Re} \nabla \cdot \boldsymbol{\tau} = D_c \rho_c \mathbf{E} \quad (13)$$

$$\frac{\partial(\rho e_0)}{\partial t} + \nabla \cdot \left[(\rho e_0 + p) \mathbf{V} - \frac{1}{Re} (\mathbf{V} \cdot \boldsymbol{\tau}) - \frac{1}{(\gamma - 1) Pr M^2 Re} \mathbf{Q} \right] = D_c \rho_c \mathbf{V} \cdot \mathbf{E} \quad (14)$$

The velocity, conduction heat transfer, and electric field vectors are defined as

$$\mathbf{V} = \begin{bmatrix} u \\ v \\ w \end{bmatrix} \quad \mathbf{Q} = \begin{bmatrix} Q_x \\ Q_y \\ Q_z \end{bmatrix} \quad \mathbf{E} = \begin{bmatrix} E_x \\ E_y \\ E_z \end{bmatrix} \quad (15)$$

The right-hand side of the equations 13 and 14 represent source terms for the dielectric barrier discharge actuator to be discussed in section II.C. The non-dimensionalization is performed using the following definitions where * represents non-dimensional quantities. Except equations 16a and 16b, the * has been dropped for simplicity and all quantities are non-dimensional in this paper, unless stated otherwise.

$$t^* = \frac{t V_{ref}}{L} \quad \rho^* = \frac{\rho}{\rho_{ref}} \quad p^* = \frac{p}{\rho_{ref} V_{ref}^2} \quad T^* = \frac{T}{T_{ref}} \quad (16a)$$

$$\mu^* = \frac{\mu}{\mu_{ref}} \quad u^* = \frac{u}{V_{ref}} \quad v^* = \frac{v}{V_{ref}} \quad w^* = \frac{w}{V_{ref}} \quad (16b)$$

The reference conditions are the freestream conditions and the length scale, L , is the same as the boundary-layer thickness, both are discussed later in section III.B. The non-dimensional parameters Reynolds number, Prandtl number, and Mach number are defined below. The specific heat at constant pressure is C_p and κ_{ref} is the thermal conductivity constant. The molecular viscosity, μ_{ref} is calculated using the Sutherland's law and a perfect gas is assumed.

$$Re = \frac{\rho_{ref} V_{ref} L}{\mu_{ref}} \quad Pr = \frac{\mu_{ref} C_p}{\kappa_{ref}} \quad M = \frac{V_{ref}}{\sqrt{\frac{\gamma p_{ref}}{\rho_{ref}}}} \quad (17)$$

The above Navier-Stokes equations can be expressed in flux-vector form as

$$\frac{\partial \mathbf{U}}{\partial t} + \frac{\partial \mathbf{F}_{inv}}{\partial x} + \frac{\partial \mathbf{G}_{inv}}{\partial y} + \frac{\partial \mathbf{H}_{inv}}{\partial z} = \frac{\partial \mathbf{F}_{vis}}{\partial x} + \frac{\partial \mathbf{G}_{vis}}{\partial y} + \frac{\partial \mathbf{H}_{vis}}{\partial z} + \mathbf{S} \quad (18)$$

where \mathbf{U} is the solution vector defined as $\mathbf{U} = [\rho, \rho u, \rho v, \rho w, \rho e_0]^{tr}$, the inviscid and viscous flux vectors are defined as

$$\mathbf{F}_{inv} = \begin{bmatrix} \rho u \\ \rho u + p \\ \rho uv \\ \rho uw \\ (\rho e_0 + p)u \end{bmatrix} \quad \mathbf{F}_{vis} = \frac{1}{Re} \begin{bmatrix} 0 \\ \tau_{xx} \\ \tau_{xy} \\ \tau_{xz} \\ (u\tau_{xx} + v\tau_{xy} + w\tau_{xz}) + \frac{1}{(\gamma-1)PrM^2} \frac{\partial T}{\partial x} \end{bmatrix} \quad (19)$$

The above equations are transformed into curvilinear coordinates and using the strong conservation form the following is obtained^{20,21}

$$\frac{\partial \hat{\mathbf{U}}}{\partial t} + \frac{\partial \hat{\mathbf{F}}_{inv}}{\partial \xi} + \frac{\partial \hat{\mathbf{G}}_{inv}}{\partial \eta} + \frac{\partial \hat{\mathbf{H}}_{inv}}{\partial \zeta} = \frac{\partial \hat{\mathbf{F}}_{vis}}{\partial \xi} + \frac{\partial \hat{\mathbf{G}}_{vis}}{\partial \eta} + \frac{\partial \hat{\mathbf{H}}_{vis}}{\partial \zeta} + \hat{\mathbf{S}} \quad (20)$$

such that $\hat{\mathbf{U}} = \mathbf{U}/\mathbf{J}$ and $\hat{\mathbf{S}} = \mathbf{S}/\mathbf{J}$ where \mathbf{J} represents the Jacobian of the transformation. The transformed flux vectors are defined as

$$\hat{\mathbf{F}}_{inv} = \frac{1}{\mathbf{J}}(\xi_x \mathbf{F}_{inv} + \eta_y \mathbf{G}_{inv} + \zeta_z \mathbf{H}_{inv}) \quad (21)$$

$$\hat{\mathbf{F}}_{vis} = \frac{1}{\mathbf{J}}(\xi_x \mathbf{F}_{vis} + \eta_y \mathbf{G}_{vis} + \zeta_z \mathbf{H}_{vis}) \quad (22)$$

and similarly $\hat{\mathbf{G}}_{inv}$, $\hat{\mathbf{H}}_{inv}$, $\hat{\mathbf{G}}_{vis}$, and $\hat{\mathbf{H}}_{vis}$.

II.C. Counterflow Force Model

The counterflow force model, which models the effect of dielectric barrier discharge (DBD) actuator, is used to trip the laminar boundary layer to turbulent. The model was developed by Shyy et al.²² and later used by Gaitonde et al.²³ in a subsonic flow control application. Mullenix et al.^{24,25} and Waindim et al.^{26,27} used it to generate supersonic turbulent boundary layers on a flat plate for use as the inflow to shock wave/boundary-layer interaction problems.

The source terms, on the right-hand side of the Navier-Stokes equations, represent the effect of a DBD actuator which is composed of a submerged electrode excited with 5kV of AC at 5 kHz as described by Shyy et al.²² The model development was rooted in experimental work and thus the model is empirical in nature. The original intent of the model was to simulate flow control in numerical calculations, however it is employed as a tripping mechanism to obtain a turbulent boundary layer in the present work.

One of the key parameters employed in the above model is the counterflow force coefficient, defined as the ratio of electrical and inertial forces.

$$D_c = \frac{\rho_c e_c E L_{ref}}{\rho_{ref} U_{ref}^2} \quad (23)$$

Here ρ_c is the discharge density and e_c is the electronic charge. The counterflow force is related to the coefficient as

$$\mathbf{F}_e = D_c \theta_f \vartheta \alpha \rho_c \mathbf{E} \delta_{cr} \quad (24)$$

Here θ_f is the frequency of the applied voltage, ϑ is the duty cycle to recover an effective mean force, α is a factor of collision efficiency, and δ_{cr} is a function of value one where the local electric field exceeds the breakdown electric field and zero elsewhere. The magnitude of the electric field vector varies linearly as

$$|\mathbf{E}| = E_0 + k_1 x + k_2 y \quad (25)$$

$E_0 = V_e/d$, where V_e is the applied voltage and d is the streamwise distance between two electrodes. Thus, the formulation allows for a peak value in the near-wall region and diminishing to zero at a specified distance away from the wall. The constants k_1 and k_2 can be calculated using a and b , the extents of the force field in y and x direction respectively. The various parameters discussed above are listed in table 1 and remain unchanged from the work of Mullenix et al.²⁴

$$k_1 = \frac{1 - E_0}{b} \quad k_2 = \frac{1 - E_0}{a} \quad (26)$$

Table 1. Counterflow force model parameters

Parameter	Value
D_c	25.0
θ_f	3000.0
ϑ	67×10^{-6}
α	1.0
ρ_c	1.0
E_0	0.3125
a	0.03125
b	0.125

III. Simulation Methodology

III.A. Numerical Schemes

A fifth-order bandwidth- and order-optimized weighted essentially non-oscillatory (WENO) scheme²⁸ was used with Roe’s formulation to compute the inviscid fluxes. The viscous fluxes were calculated using a sixth-order compact²⁹ spectral-like scheme. The implicit time integration was performed with the second-order Beam-Warming scheme using two sub-iterations and approximate factorization. The non-dimensional time step for this problem was 0.002.

III.B. Boundary Conditions and Mesh Parameters

The flow conditions represent the experiments³⁰ performed at the Ohio State University’s Gas Dynamics and Turbulence Lab. Table 2 shows a comparison of experimental and simulated flow conditions. It is noted that to resolve the scales of turbulence corresponding to the experimental Reynolds number, a mesh various orders of magnitude larger than the fine mesh used here would be required. Thus, the simulated Reynolds number was lowered to be practical and achievable by ILES. Such an approach was also taken by Mullenix and Gaitonde,²⁵ Touber and Sandham,³¹ and Visbal et al.³² In order to have consistent flow relations, the pressure was reduced by a factor of 10 while maintaining the temperature. This assures that the Mach number and the velocity scales remain the same between the experiment and the simulation. This was done so that the frequencies associated with the velocity scales remain comparable between the experiment and the simulation. It will be discussed later that the reduction of Reynolds number by a factor of 10 might not be sufficient and further scaling is necessary for the present flow conditions and meshes.

Table 2. Flow conditions

Property	Experiment	Simulation
M_∞	2.33	2.33
U_∞ , m/s	556.0	556.0
P_∞ , Pa	23,511.0	2351.1
T_0 , K	295.6	295.6
T_w , K	269.7	269.7
δ_{99} , m	5.3×10^{-3}	5.3×10^{-3}
Re_δ	175,202.0	17,520.2

At the inflow, a laminar boundary layer profile, calculated using a compressible Blasius solution was imposed. The interior of the flowfield was also initialized with the same Blasius solution. The wall was treated as no-slip and the ratio of expected adiabatic wall temperature to freestream static temperature was 1.95. The outflow and farfield boundaries were obtained by extrapolating values from the interior. A periodic boundary condition was applied in the z direction. The Rankine-Hugoniot relations were used to impose the oblique shock generated by a wedge in the experiment. Such a simulated shock was achieved by

maintaining the farfield boundary condition for $x < x_{shock}$ at the pre-shock conditions identified in table 2, while $x > x_{shock}$ were set to conditions obtained via oblique shock relations at Mach 2.33 and angle of attack set to 9 degrees. The shock location, x_{shock} , was set to 23.19 to obtain the inviscid shock impingement location of $x \sim 62.0$.

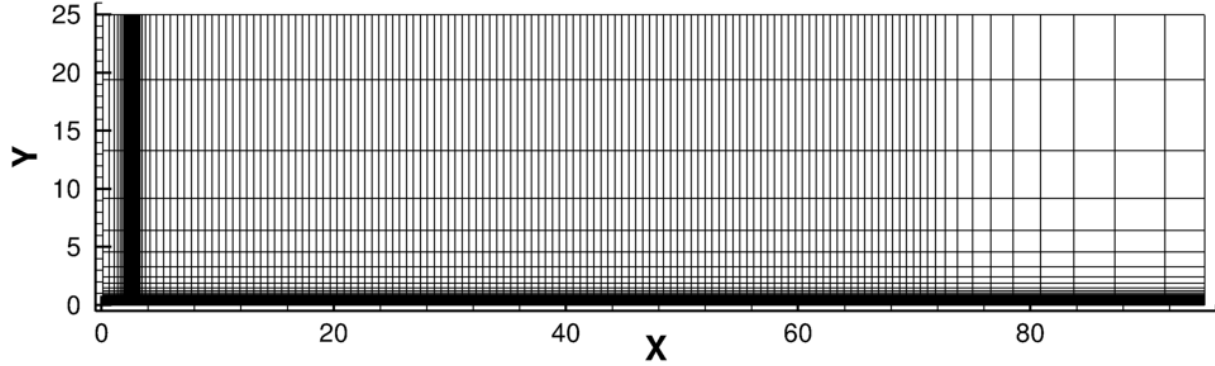


Figure 2. Mesh topology for SBLI simulation

Table 3. Mesh parameters

	Coarse	Medium	Fine
Domain size			
$x \times y \times z^a$	$95 \times 25 \times 5$	$95 \times 25 \times 5$	$95 \times 25 \times 5$
Computational points			
$N_x \times N_y \times N_z$	$917 \times 167 \times 133$	$1101 \times 201 \times 167$	$1301 \times 251 \times 201$
N_{total}	2.025×10^7	3.696×10^7	6.564×10^7
Counterflow force region			
N_x	100	100	100
x_{center}	2.6	2.6	2.6
x_{length}	1.0	1.0	1.0
Δx	0.010	0.010	0.010
Constant region			
N_x	666	833	1000
Δx	0.0997	0.0846	0.0665
$\Delta x_{x=65}^+$	23.51	20.55	16.35
$N_{y,bl}$	101	125	151
Δy_w	1.96×10^{-3}	1.96×10^{-3}	1.96×10^{-3}
$\Delta y_{x=65}^+$	0.4621	0.4764	0.4822
Δz	0.0376	0.0299	0.0250
$\Delta z_{x=65}^+$	8.86	7.26	6.15

^a x, y, z are non-dimensionalized by δ_{99}

In figure 2, the mesh schematic is presented where the coordinates are non-dimentionalized by the length scale L . For the purpose of clarity, every fourth point is shown. The mesh can be divided into five distinct sections: 1) the inflow section where the mesh is refined until it reaches the counterflow force section, 2) the counterflow force section where a constant axial spacing is maintained with the actuator located at $x = 2.6$, 3) a coarsening section, 4) the constant spacing section is where the SBLI occurs, and 5) the coarsening to outflow section. All sections have the same hyperbolic tangent grid clustering in the y direction and a constant spacing in the z direction. Table 3 shows a list of parameters for coarse, medium, and fine

meshes. Although all three meshes were investigated, only the fine mesh results are presented here. The mesh sensitivity study of Mullenix and Gaitonde²⁵ was the basis for this decision.

IV. Results

In this section, the validation of the upstream boundary layer is presented first and followed by detailed turbulent kinetic energy budgets of the SBLI region.

IV.A. Validation of upstream boundary layer

In order to obtain a qualitative description of the developing turbulent boundary layer, the Q -criterion, as described by Jeong and Hussain,³³ was computed and the iso-surfaces of $Q = 0.5$ are presented in figure 3 colored by \bar{u} velocity. The effect of the counterflow force actuator can be seen at the beginning of the computational domain followed by the laminar-to-turbulent transition phase, which results in coherent structures. Among the coherent structures are hairpin vortices, figure 4, peaking from the smaller structures in the boundary layer. The change in the boundary layer is noticeable beyond the shock wave/boundary-layer interaction. Towards the end of the computational domain the structures disappear due to grid coarsening. In a mesh refinement study, Mullenix et al.²⁴ has shown a similar behavior at $Q = 1.0$ for a variety of meshes, where it was also noted that refining of the mesh led to a higher concentration of resolved coherent structures and a quicker transition process.

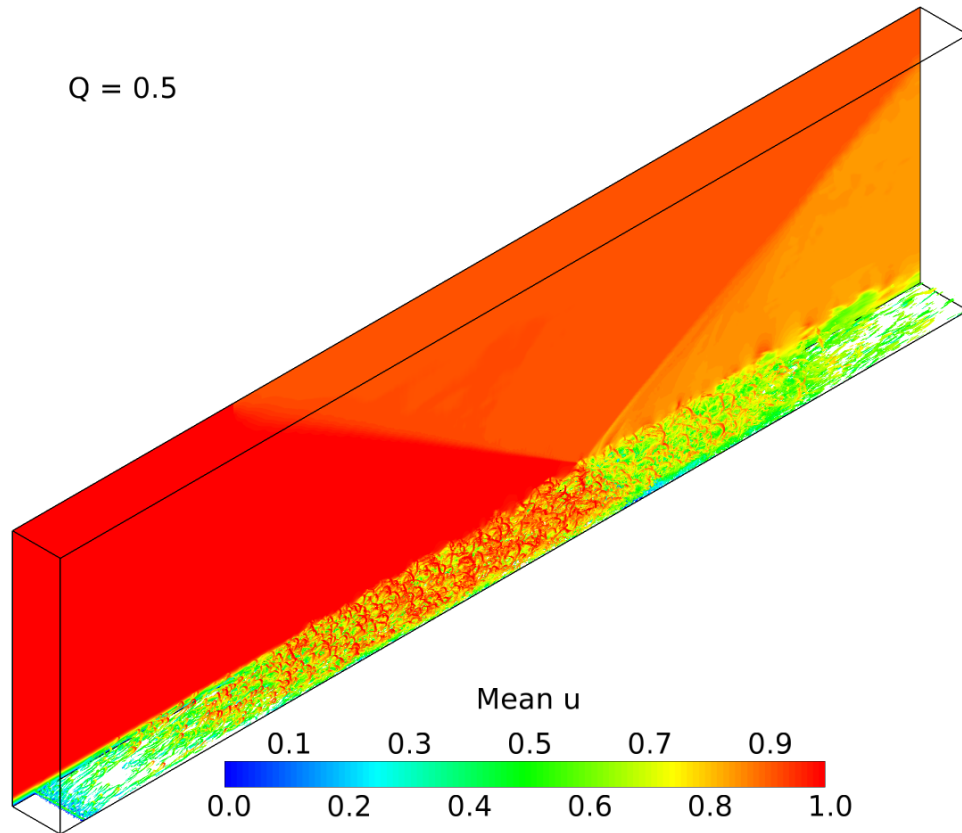


Figure 3. Iso-surface of Q -criterion at $Q = 0.5$ colored by \bar{u} velocity

The remainder of the results that follow are time averaged for six flow-through times. The time-averaged results were compared three flow-through times apart to ascertain statistical convergence of mean quantities and agrees with observations of Bisek,³⁴ where a similar approach was used to determine convergence. The skin friction through the laminar-to-turbulent boundary layer transition is presented in figure 5(a) along with the theoretical^{35,36} laminar and turbulent flat plate skin friction profiles. It is clear that the centerline skin

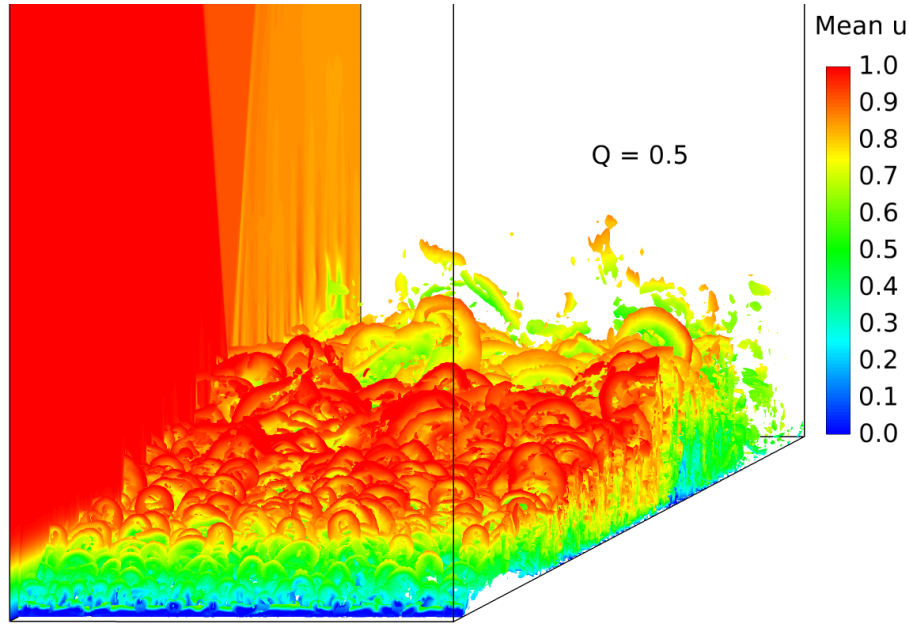


Figure 4. Hairpin vortices, Iso-surface of Q-criterion at $Q = 0.5$ colored by \bar{u} velocity

friction coefficient oscillates about the theoretical turbulent curve beyond $x = 60$, however the span-averaged skin friction was marginally lower. It is noted that in the work of Bisek³⁴ a similar comparison was presented for three mesh levels, coarse, medium, and fine at approximately the same flow conditions. As the mesh was refined, a quicker transition to turbulent skin friction and a better match with the theoretical curve was obtained. Since the fine mesh results presented here are obtained with half the coarse mesh resolution used by Bisek,³⁴ it is concluded that a finer mesh would provide an improvement over the current prediction, but at a significant additional computational cost. A comparison of the centerline and span-averaged velocity profiles with the inner layer and the logarithmic law of the wall is provided in figure 5(b). The velocity was transformed to the incompressible form using the van Driest transformation.³⁷ Both the centerline and span-averaged profiles at $Re_\theta = 3270$ show the expected behavior of the turbulent boundary layer and agree with the highly-resolved (3 billion point) incompressible DNS simulation of Schlatter and Örlü.³⁸ The value $Re_\theta = 3270$ represents $x = 56.0$, the station upstream of the area where the SBLI, to be discussed in section IV.B, occurs. Thus, this location was an important landmark to validate the developed boundary layer before the SBLI region.

Next, the Reynolds shear stress is compared with the incompressible DNS of Schlatter and Örlü³⁸ in figure 6 for $Re_\theta = 3270$ and 3630 . Both DNS profiles show self-similarity and are consistent with the expected behavior of Reynolds shear stress for a fully developed equilibrium boundary layer. The predicted profiles in the present simulation for both Re_θ show a trend where Reynolds shear stress is underpredicted when normalized by friction velocity (u_τ^2) and overpredicted when the normalized Reynolds shear stress is scaled by local density ($\bar{\rho}/\bar{\rho}_w$). Also, the Reynolds shear stress at $Re_\theta = 3270$ peaks above 1.0 in the log layer. This is a concern for two potential reasons: 1) it is a sign of inadequate resolution at the boundary layer edge and 2) the boundary layer has not reached an equilibrium state at that station. This was also observed by Gatski and Erlebacher³⁹ and Maeder et al.,⁴⁰ although their work was inconclusive on the cause of this behavior. However, Guarini et al.¹⁸ and Pirozzoli et al.⁴¹ later showed with their highly resolved Mach 2.25 and 2.5 DNS simulations, respectively, that the compressible flow simulations should follow established incompressible flow data as proposed by Morkovin,⁴² albeit density scaling was necessary as shown by Guarini et al.¹⁸ It is important to note that the predicted Reynolds shear stress at $Re_\theta = 3630$ matches better with the DNS suggesting that the boundary layer is approaching an equilibrium state albeit at a longer axial distance. Moreover, it was found that in the viscous sublayer $-\bar{uv}^+ = 0.0008y^+$ for both Re_θ discussed above, which was also shown by Patel et al.,⁴³ where the authors made a thorough comparison of various turbulence models with experimental data.

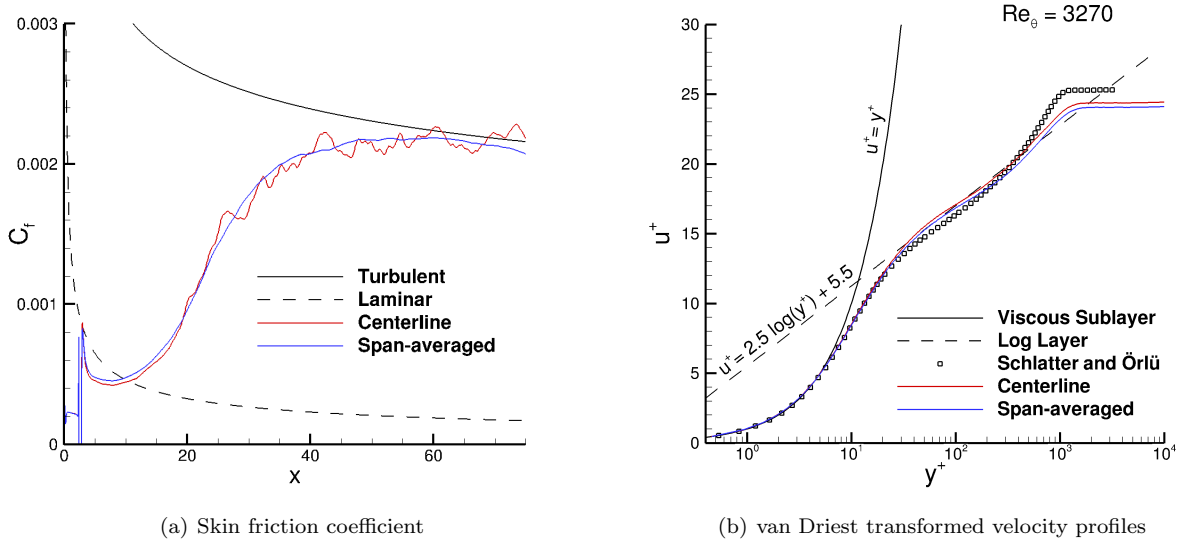


Figure 5. Centerline and span-averaged characteristics of the upstream boundary layer

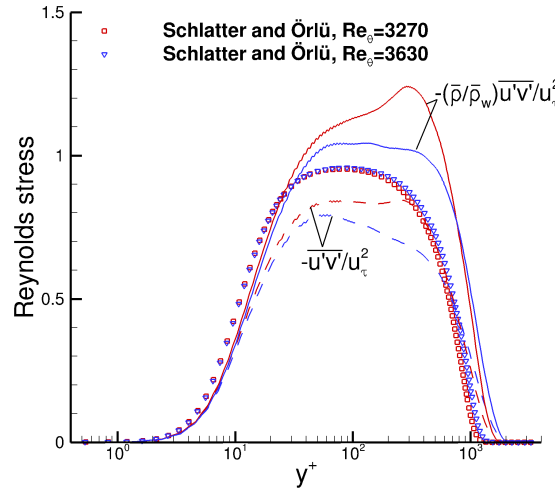


Figure 6. Span-averaged Reynolds shear stress at $Re_\theta = 3270$ (red) and 3630 (blue)

The turbulent kinetic energy and Reynolds normal stresses show a similar behavior as the Reynolds shear stress. Compared to the incompressible DNS of Schlatter and Örlü,³⁸ figure 7(a), $\overline{v'v'}$ and $\overline{w'w'}$ are underpredicted in the range of $10 \lesssim y^+ \lesssim 100$, while $\overline{u'u'}$ was overpredicted. Beyond $y^+ \sim 100$, $\overline{v'v'}$ and $\overline{w'w'}$ are closer to the DNS, yet underpredicted and $\overline{u'u'}$, contrary to the buffer layer region, was underpredicted. As discussed before, Pirozzoli et al.⁴¹ and Guarini et al.¹⁸ have shown that scaling the Reynolds stress by local density improves the comparison of compressible predictions with incompressible DNS. However, as shown in figure 7(b), improvement was only realized in the $\overline{v'v'}$ and $\overline{w'w'}$ components for the buffer layer and beginning of the log layer regions. Density scaling does however show that $\overline{u'u'}$ was consistently overpredicted rather than what was observed for the unscaled comparison in figure 7(a).

Finally, the budget of turbulent kinetic energy is presented in figure 8 to validate the profiles forward of the SBLI region. The budget terms are normalized by $\bar{\rho}_w u_\tau^4 / \nu_w$. The production, dissipation, molecular diffusion, and turbulent transport are compared with Schlatter and Örlü³⁸ and presented at a matching Re_θ . Although, the present simulation follows the trend of the incompressible DNS, the magnitude of the terms

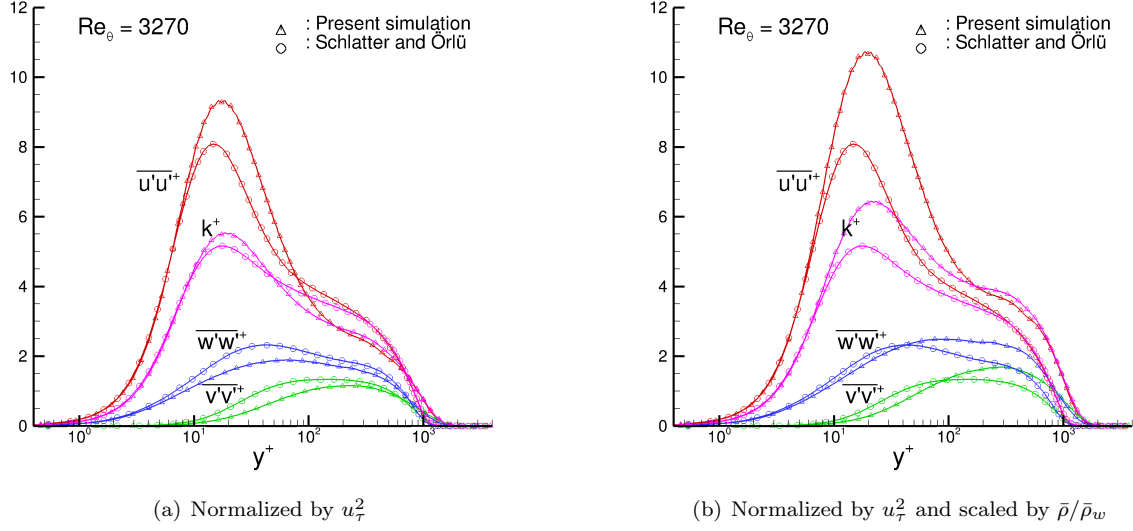


Figure 7. Span-averaged turbulent kinetic energy and Reynolds normal stress

in the budget are larger. This can be attributed to the lack of mesh resolution at the simulated Reynolds number (cf. table 2), suggesting that a finer mesh or lowering the simulated Reynolds number may improve comparison with the DNS.

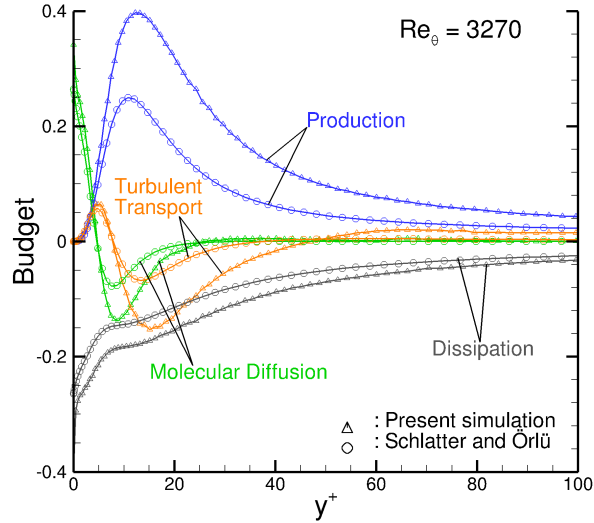


Figure 8. Span-averaged budget of the turbulent kinetic energy at $Re_\theta = 3270$

IV.B. Budgets for Shock Wave/Boundary-layer Interaction

The analysis of the shock wave/boundary-layer interaction (SBLI) will focus on budgets of the turbulent kinetic energy at key locations in the SBLI region. Comparisons of the mean flow and separation characteristics with the experimental³⁰ measurements are omitted here and readers are directed to the work of Mullenix and Gaitonde,²⁵ where successes and shortfalls of the predictions using the present methodology were discussed.

Figure 9 shows contours of \bar{u} in the vicinity of the SBLI at the spanwise centerline and identifies key locations, listed in table 4, which will be used during the discussion of the results. The dashed lines mark

theoretical extension, to the wall, of the mean impinging and the reflected shocks. Under the confluence of these shocks, a separation bubble is evident in the interaction region. A comparison of the centerline and span-averaged skin friction, presented in figure 10, show that between $x = 58.2$ and $x = 60.9$, the skin friction becomes negative—an indication of flow separation.

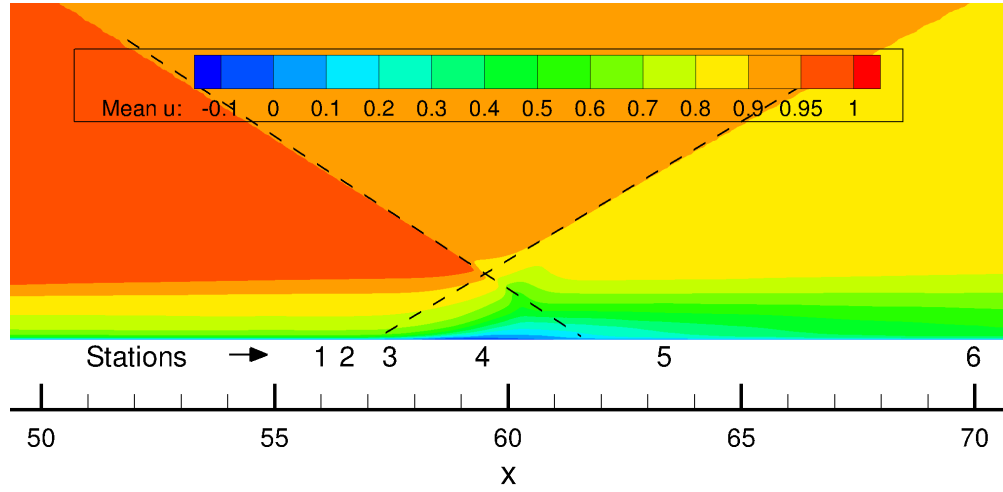


Figure 9. Contours of \bar{u} at the domain centerline

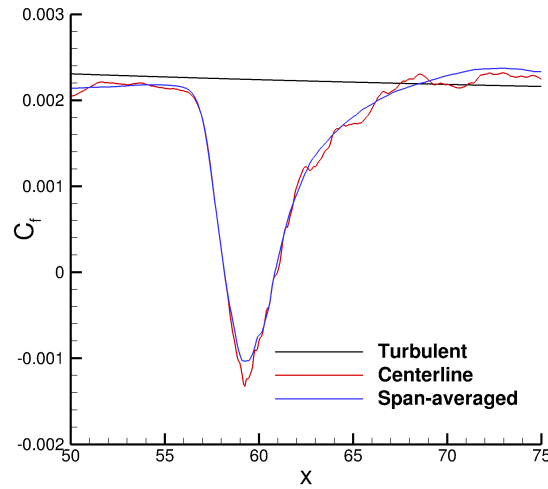


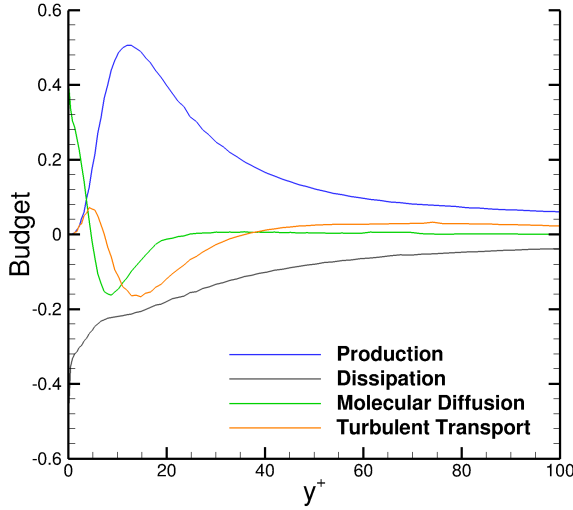
Figure 10. Skin friction in the SBLI region

Table 4 shows a list of key locations for which the budget will be presented and the corresponding Re_θ where applicable. The positions are relative to the mean impinging and reflected shocks and defined in figure 9.

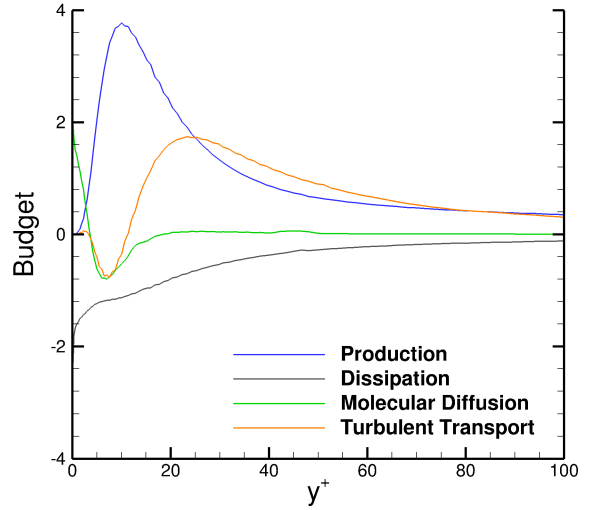
The budgets are presented to indicate the behavior of the terms as the flow approaches the interaction region, within the interaction, and after. In all plots, production, dissipation, molecular diffusion, and turbulent transport terms are shown. As before, the budget terms are normalized by $\bar{\rho}_w u_\tau^4 / \nu_w$. The pressure diffusion, pressure dilatation, and mass flux terms were small such that they would appear as zero on each plot, thus they are omitted.

Table 4. Key locations

Station	Location	x	Re_θ
1	Incoming flat plate boundary layer	56.0	3270
2	Upstream of the reflected shock	56.6	3280
3	Downstream of the reflected shock	57.5	3300
4	Separation bubble	59.5	-
5	Downstream of the impinging shock	63.4	-
6	Recovered flat plate boundary layer	70.0	-

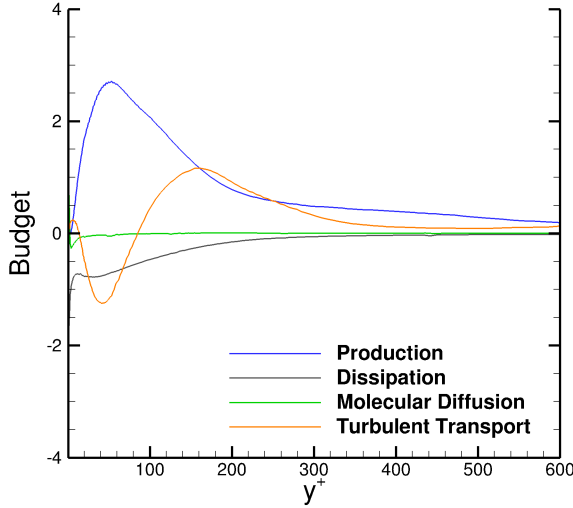


(a) Upstream, $x = 56.6$ and $Re_\theta = 3280$

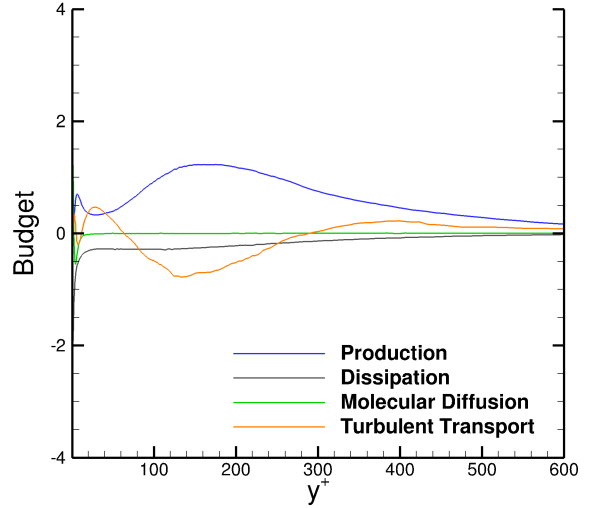


(b) Downstream, $x = 57.5$ and $Re_\theta = 3300$

Figure 11. Span-averaged budget upstream and downstream of the mean reflected shock



(a) Separation bubble, $x = 59.5$



(b) Relaxation region, $x = 63.4$

Figure 12. Span-averaged budget upstream and downstream of the mean impinging shock

The effect of the interaction region was observed as early as station $x = 56.6$ (figure 11(a)), where a marginal increase in the budget terms can be observed (cf. with figure 8). The increase in magnitude for all four quantities was evident at station $x = 57.5$, which was just aft of the extension, to the wall, of the

mean reflected shock position. Particularly, the production and dissipation terms increased by an order of magnitude. The turbulent transport term (cf. figures 11(a) and 11(b)) shows that the trough in the buffer layer ($y^+ = 14.5$) has moved closer to the wall and a new peak has developed at the beginning of the log layer. This fundamentally alters the near wall dynamics responsible for transfer and propagation of the turbulent kinetic energy. Interestingly, at this station, peaks and troughs for production, molecular diffusion, and dissipation terms remain at approximately the same location in the wall normal direction as the station forward of the reflected shock.

Figure 12 shows two locations, station $x = 59.5$ within the separation bubble and $x = 63.4$ past the theoretical extension, to the wall, of the mean impinging shock. Note that the abscissa has been extended to $y^+ = 600$ to accommodate the active region of the boundary layer. In the separation bubble, figure 12(a), the peak in the production term moved away from the wall, out of the buffer layer, and into the beginning of the log layer. The shape of the peak broadened and spanned up to $y^+ = 200$, well into the log layer. The turbulent transport term also shifted away from the wall with a peak at $y^+ = 170$, showing a coupling between the production and the turbulent transport terms. Also the peak in the sublayer has grown in magnitude for the turbulent transport term, a sign of activity in the near wall region. The molecular diffusion and dissipation terms showed no change in dynamics, but did show a change in magnitude. In the relaxation region, figure 12(b), a lower peak in the production term, reminiscent of the boundary layer before the reflected shock has developed at $y^+ = 6.5$, an indication of the flow returning back to the undisturbed boundary layer state. Also, a larger peak at $y^+ = 150$ was also observed which corresponds to the high production region evident in figure 15. The turbulent transport term behavior also showed near wall behavior similar to the undisturbed boundary layer, a peak in the sublayer and a trough in the buffer layer. A trough near $y^+ = 150$ seems to coincide with a peak in the production at the same location.

Further downstream, the boundary layer begins to approach an equilibrium form and the budgets, presented in figure 13, showed identical near wall behavior to that of station $x = 56.0$, but with marginally higher magnitudes. Although not shown here, the production and turbulent transport terms remained active in the log and outer layer regions ($10^2 < y^+ < 10^3$) with secondary peaks and troughs.

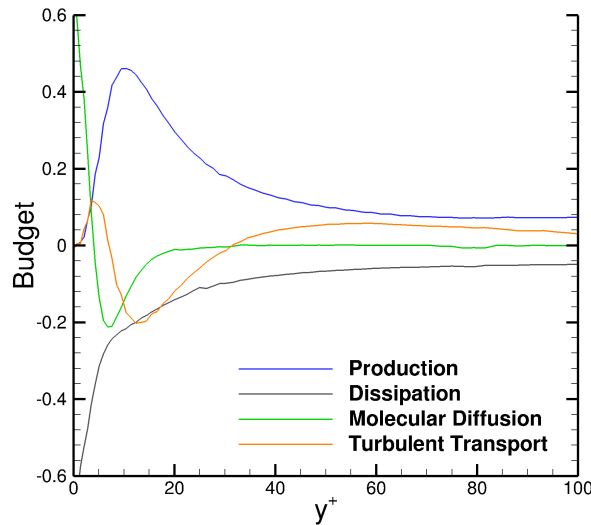


Figure 13. Span-averaged budget of the turbulent kinetic energy at $x = 70.0$

The ratio of production-to-dissipation of the turbulent kinetic energy, \mathcal{P}/ϵ , can be often found in the literature pertaining to turbulence modeling.^{43,44} In wall-bounded flows, it can be used to confirm a local equilibrium in the log layer of a turbulent boundary layer. In figure 14, \mathcal{P}/ϵ is plotted for the incompressible DNS of Schlatter and Örlü³⁸ along with various stations of the present simulation. The DNS data shows that in the log layer the ratio approach the value of unity as expected, an indication of local equilibrium. A comparison of the incoming turbulent boundary layer at station $x = 56.0$ ($Re_\theta = 3270$), upstream of the SBLI region, shows that the present simulation does not attain a complete equilibrium form in the log layer. Nonetheless, the profiles at two upstream stations ($x = 56.0$ and $x = 56.6$) and one downstream station

($x = 70.0$) matched reasonably well with the DNS in the viscous sublayer and exhibits self-similarity. In the SBLI region, stations $x = 57.5$ and $x = 59.5$, a dramatic change in the ratio is evident and the local flowfield is far from equilibrium as expected.

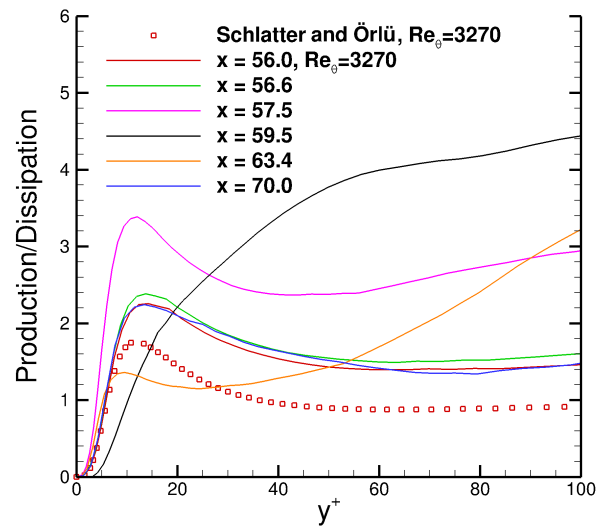


Figure 14. The production-to-dissipation ratio, \mathcal{P}/ϵ

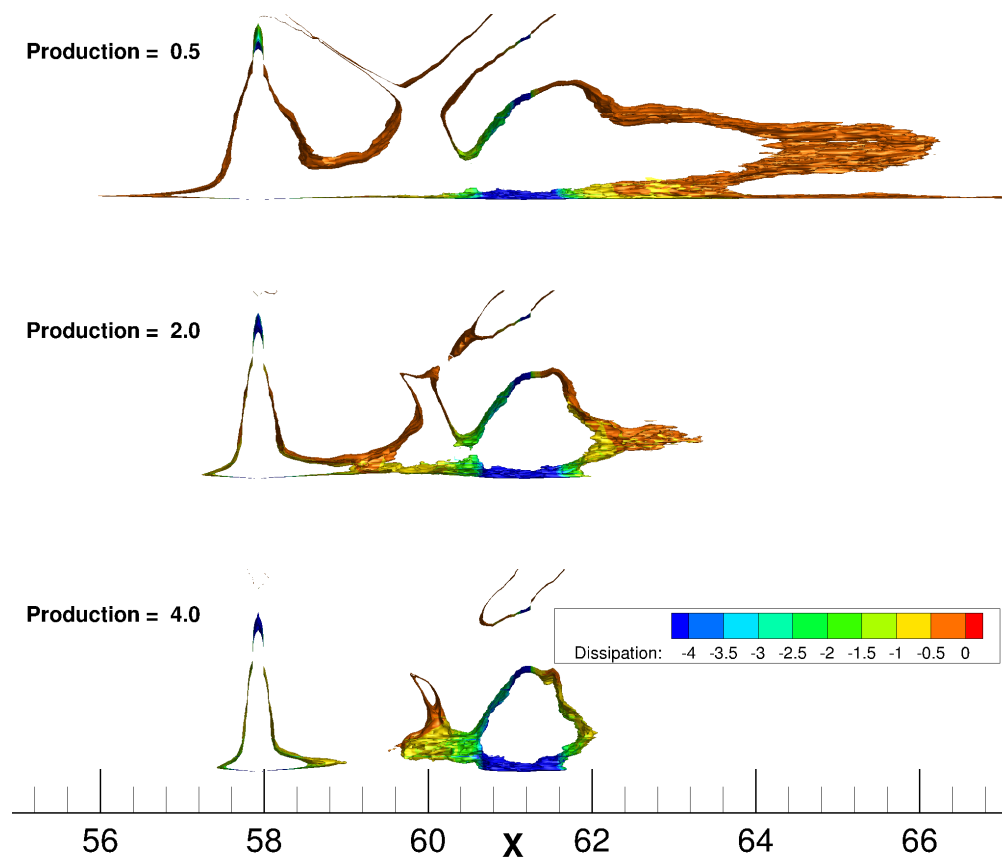


Figure 15. Iso-surface of the production term at 0.5, 2.0, and 4.0 and colored by dissipation

Iso-surfaces of the production term are presented in figure 15 at three levels, 0.5, 2.0, and 4.0. They are colored by dissipation to show regions where dissipation may balance production of the turbulent kinetic energy for each iso-surface. The values are normalized by $\bar{\rho}_w u_\tau^4 / \nu_w$. The amplification of the production term as flow approaches the SBLI is clear. The large production rate around the mean impinging and reflected shocks was local and the production drops quickly away from the shocks. The region between the two shocks show large magnitude of production, which is not balanced by dissipation. The production in the relaxation region continues at elevated levels and subsides gradually downstream.

V. Conclusion

The budget for turbulent kinetic energy has been presented forward, through, and aft of an SBLI. Validation of the ILES approach by means of comparison with the DNS showed that the fine mesh considered in the current simulations was inadequate to accurately resolve the turbulent scales in the buffer layer and log layer even at a reduced Reynolds number. However, the current simulation predicted matching trends with that of the DNS when scaled by local density.

Comparison with the DNS also suggested that the boundary layer may not have reached a complete equilibrium forward of the SBLI, thus a longer axial domain for the boundary layer development, aft of the counterflow force actuator, is necessary. Alternatively, newer methods of specifying unsteady inflow conditions are being investigated.

The budgets presented for the SBLI provide insight into the dynamic nature of the interaction region, i.e., the quickly changing nature of the production and the turbulent transport terms and more subdued nature of the dissipation and the viscous diffusion terms. In particular, the turbulent transport term showed shifting of peaks and troughs in the wall normal direction. A new peak also developed immediately aft of the reflected shock suggesting a jump-start in the transport process to compensate for the increase in the production term. Large magnitude of the budget terms persisted in the interaction region, but began to subside in the relaxation region. At the farthest downstream location the budget looked near identical to that of the incoming boundary layer upstream of the SBLI. Overall, considering the difference in mesh resolution between the present ILES versus the DNS, the ILES predictions captured the trends reasonably well.

Acknowledgments

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