TFAWS Passive Thermal Paper Session

Review of MLI Behavior at Low Temperatures and Application to L'Ralph Thermal Modeling Daniel Bae Juan Rodriguez-Ruiz

> Presented By Daniel Bae

ANALYSIS WORKSHOP

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Outline

- Introduction
- MLI Performance Behavior
- Literature
- Application to TD
- Additional Topics: IMLI, Multi-netting, Silk



Why Study MLI?



- Published ϵ^* values vary wildly
- ϵ^* values depend on temperature and L'Ralph has wide range of temperatures
 - 100 K (IR Detector), 180 K (Vis Detector), and 300 K (Main Electronic Box)
 - Predecessors to L'Ralph are running slightly warmer than expected



1) Multi-layer insulation (MLI) (208 mW)

- 2) Mechanical Supports (90 mW)
- 3) Electrical Harness Parasitics (49 mW)
- 4) IR detector radiative exchange with optics bench interior (35 mW)
- 5) Backloads from L'Ralph's external surfaces (18 mW)



MLI ϵ^* Sensitivity





MLI Behavior



• ϵ^* can be used to represent total MLI behavior as only a "radiation"

$$- q'' = G_{LIN}(T) \cdot (T_H - T_C) + G_{RAD}(T) \cdot \sigma \cdot (T_H^4 - T_C^4) = \epsilon^*(T) \cdot \sigma \cdot (T_H^4 - T_C^4)$$

• If MLI was conduction dominant (i.e. $G_{RAD}(T) \cong 0$) and we were representing the overall MLI effectiveness with ϵ^* , we would see an increase in ϵ^* as temperature goes down due to lower order behavior of linear conductance

$$- \quad \epsilon^*(T) = \frac{G_{LIN}(T)}{\sigma} \cdot \frac{(T_H - T_C)}{(T_H^4 - T_C^4)}$$

- If we assume $G_{LIN}(T) \sim constant = 0.025 \frac{W}{m^2 \kappa}$, we would see ϵ^* behavior shown below
- The challenge is figuring out how much contribution comes from the linear and radiation terms based on the construction and design of the MLI





Blanket Performance Variables

- Emissivity of the materials
- # of blanket layers
- Compression of blanket structure (blanket density)
- Blanket size / footprint
- Thermal spacer resistance
- Gasses within the blankets
- Venting techniques
- Perforations
- # of seams
- Workmanship



STANDARD BLANKET





- Staggered seams to reduce conduction
- Better layer density; more care and more "poofy" construction

Shell method is basically "splitting" the big blanket into layers so that the ground and seam effects are smaller. This is labor intensive but does minimize the heat transfer.





MLI Effectiveness



- Keller, 1974 "Thermal Performance of Multilayer Insulations," Lockheed Martin
- Doenecke, 1993 "Survey and Evaluation of Multilayer Insulation Heat Transfer Measurements," SAE Deutsche Aerospace AG
- Johnson, 2007 "Thermal Performance of Cryogenic Multilayer Insulation at Various Layer Spacings," Auburn Univ,
- Kawasaki, 2012 "Temperature Dependence of Thermal Performance in Space Using Multilayer Insulation," JAXA
- Rodriguez-Ruiz, 2013 "MLI Effectiveness: Form Fitted, Tented and High/Low *ε*," GSFC NASA
- Harpole, 2013 "Cryo MLI Thermal Performance Correlation and Modeling," JWST, Northrop Grumman
- Nast, 2014 "Multilayer Insulation Considerations for Large Propellant Tanks," LM, NASA
- Ross, 2015 "Quantifying MLI Thermal Conduction in Cryogenic Applications from Experimental Data," JPL
- Tiedemann, 2016 "Correlation of MLI Performance Measurement with a Custom MATLAB Tool," HPS GmbH, Germany
- And more...





Various MLI Correlations

Since we like to think of MLI in terms of ϵ^* , the following correlations have be rearranged in the form of ϵ^*

Not a simple problem and many different contributors



- Assumptions:
 - Overall:
 - 13 layers double aluminized
 - Avg MLI area = 0.1 m² 0
 - 40 layers/cm
 - $\frac{T_C}{T_H} = 0.5$
 - Correlation Specific:
 - LM 1974
 - $-\epsilon_{RT}=0.033$
 - Doenecke
 - $f_N = 1.22$ - $f_A = 2.36$
 - $f_{n} = 2.30$ - $f_{n} = 1.1$
 - JAXA
 - $\quad \epsilon_{eff-R} = 0.0017$

Blanket Estar

- $H_{MLI} = 0.0062$
- $C_{Hem} = 0.016$
- NG-LM

$$- F = 8 \cdot f_{NG-LM}(T_m)$$

$$- C_A = 1.6 \cdot 10^{-5}$$

• Note that values are high because L'Ralph is a fairly small instrument



NASA



- There is a wide spread of ϵ^* between different correlations
 - Too many factors that contribute to the formulation of ϵ^*
 - Difficult to choose which correlation to use
- Use average of all the correlations for now until testing results prefer one correlation over another, and bias $\pm 33\%$ around the average
 - Allows us to capture fairly accurate ϵ^* value and behavior as a starting point
- Convert ϵ^* to K_{eff} (i.e. $K_{eff} = \epsilon^*(T) \cdot \sigma \cdot L_{MLI} \cdot \frac{(T_H^4 T_C^4)}{(T_H T_C)}$)
 - TD cannot accept temperature dependent ϵ^* (unless one modifies SINDA input manually), but allows direct temperature dependent conductivity input for insulation connection
 - Note that TD, by default, uses the average temperature between nodes for any temperature dependent conductors
 - Must input K value as a function of average, not hot or cold side temperature
 - We can also use $4T_m^3 \cong \frac{T_h^4 T_c^4}{T_h T_c}$ relation to find K_{Eff} if only $\epsilon^*(T_m)$ was available
- Create low, nominal, high ϵ^* material properties and apply them based on the insulation heat flow direction in the model
- Based on this method, the thermal model would use approximately the following ϵ^* ranges at each temperature zones
 - Note that the actual ϵ^* will be determined by the actual structure to insulation mean temperature within the simulation calculations

	Low ϵ^*	High ϵ^*
LEISA (100 K zone)	0.05	0.10
MVIC (180 K zone)	0.03	0.06
MEB (300 K zone)	0.007	0.015





Temperature



- Plots are based on the average of the correlations shown in an earlier slide
- L'Ralph cryosystem undergoes wide temperature range (between 90 180 K)
 - If non temperature dependent value was used, would not capture performance accurately at different temperature zones





- L'Ralph will be using IMLI (DAM separated by low thermal conductance polymer spacers).
 - Aside from grounding paths, edges, and seams, all conductive paths through IMLI are well determined
- Experimental heat flux closely matches modeled IMLI performance.
- Below table shows IMLI estar for 100 and 180K boundary temperatures with conservative 25% degradation allotted for penetrations, etc. Information Quest Thermal Group.

					0.0
	Heat Flux	Total			<u> </u>
# Layers	(W/m²)	estar	mass (kg)	Thickness (cm/inches)	0.0 Esta
5	0.415	0.0077	0.036	0.90/0.36	0.0
10	0.205	0.0038	0.073	1.80/0.71	0.0
20	0.101	0.0019	0.147	3.61/1.42	







Quest Discrete Spacer Insulation Family

UES

	Application	Status	TRL
Integrated MLI (IMLI)	In space, high vacuum, replaces conventional MLI	Spaceflight. Available.	9
Load Responsive MLI (LRMLI)	One atmosphere to high vacuum, replaces SOFI	Phase 3 completed	5
Load Bearing MLI (LBMLI)	Supports thermal/Broad Area Cooled shields for active cooled systems	Phase 3 completed	6
Vapor Cooled Structure MLI	Active and passive vapor cooling of tank support elements	Phase II complete	5
Multi-Environment MLI (MEMLI)	Operates in environments from space to on- Mars, ISRU surface liquefaction	In Phase II	4
Wrapped MLI (WMLI)	Cryo pipes and plumbing components	Phase II SBIR completed	5
Launch Vehicle MLI	External launch vehicle cryotanks	Phase I SBIR completed	4
Micrometeoroid and Orbital Debris IMLI	High vacuum thermal insulation and MMOD protection	Phase I SBIR completed	4
Vacuum Cellular MLI	Launch vehicles	Early dev	3
Variable Radiator	Spacecraft thermal control	Phase II SBIR in progress	4

Multi-Netted MLI and Silk vs Dacron Netting

- Multi-netting
 - Instead of single Dacron meshing between layers, multiple can be used to reduce the conductive term
- Dacron vs Silk
 - Netting switched to Dacron around 1970s due to cost of silk
 - Published papers claim that there is a significant difference between the silk and Dacron netting, with silk showing >2x better performance
 - 1974's extensive testing done by LM was done with silk netting

