

Design of P-3 Nadir Port

Early Career Forum: Structures, Loads, and Mechanical Systems

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Introduction

- Started at NASA WFF through the Pathways Program 2012-2014
- Converted to full-time Dec. 2014
- Design of science installations for flight on the WFF aircraft P-3B, C-130H, & Sherpa
- Support WFF Sounding Rockets, Internal Research and Development (IRAD), and SmallSat



Water Content Multi-element System (WCM) & Axial Cyclone Cloud-Water Collector (AC3) Pylon for Cloud, Aerosol and Monsoon Processes Philippines Experiment (CAMP²Ex)

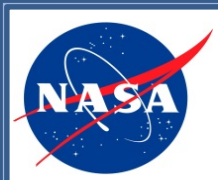
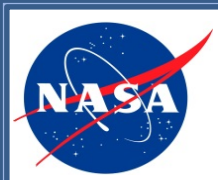


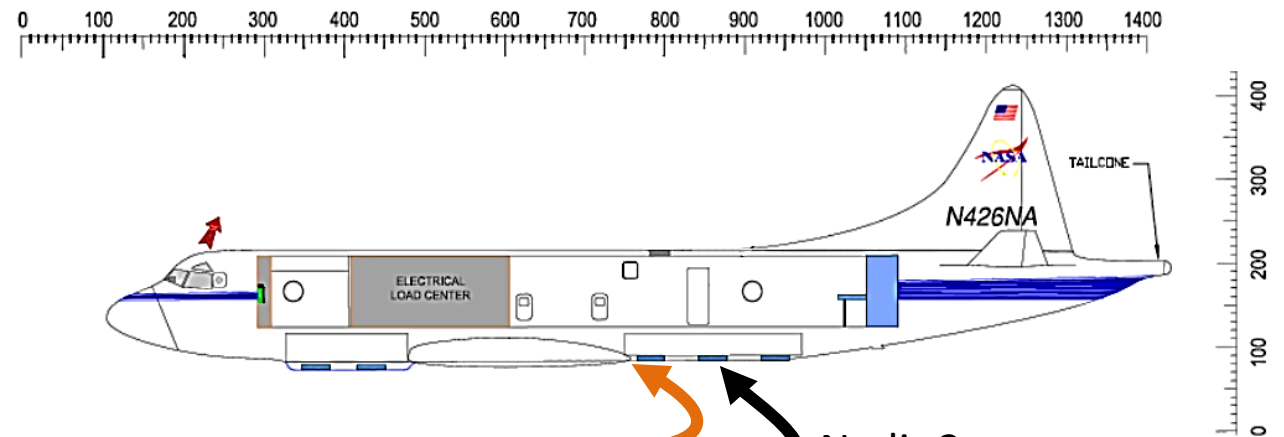
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P-3B Introduction



Nadir 1

Nadir 2

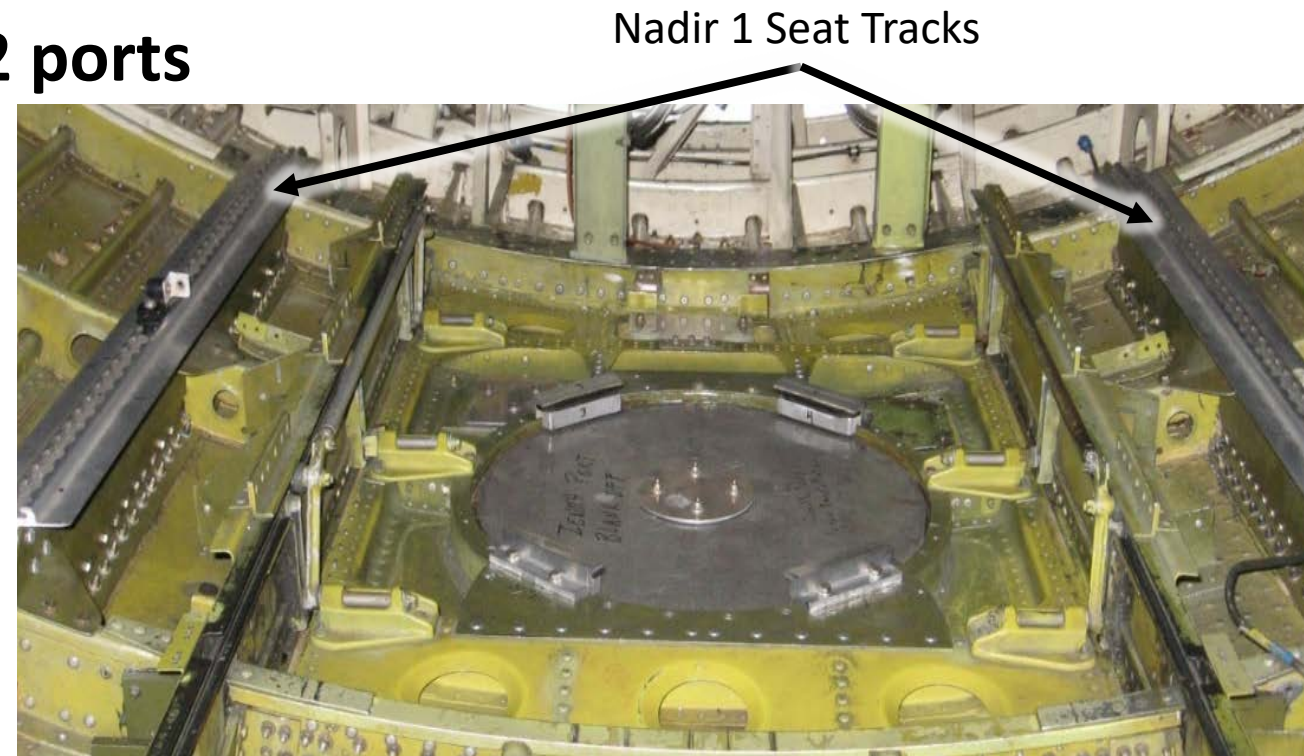
NASA GSFC WFF P-3B ORION LAYOUT



Standard Sonobuoy Door Install

Project Goals and Objectives

- WFF Aircraft Project Office (Code 830) requirements:
 - Use existing aircraft interface
 - Interchangeable instrument/port/window structures for mission-specific instruments
 - **Must fit both Nadir 1 and Nadir 2 ports**
 - Nadir 2 does not have the seat tracks
 - **Must maintain removability of port**



Internal view of door install in Nadir 1

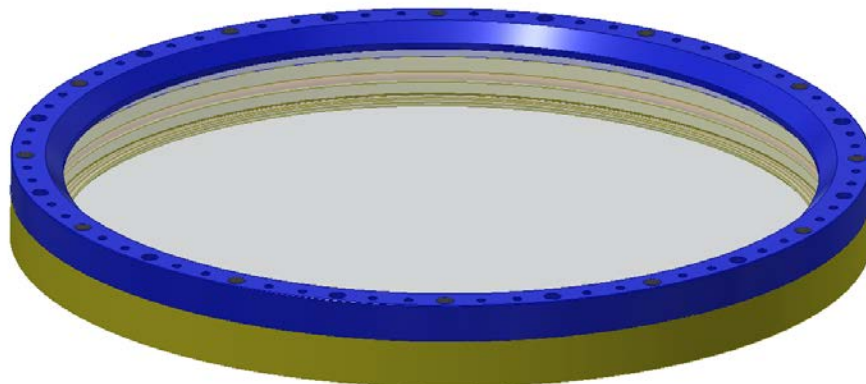
Project Goals and Objectives

- Science requirements, Operation Ice Bridge (OIB):
 - Support the following window specs

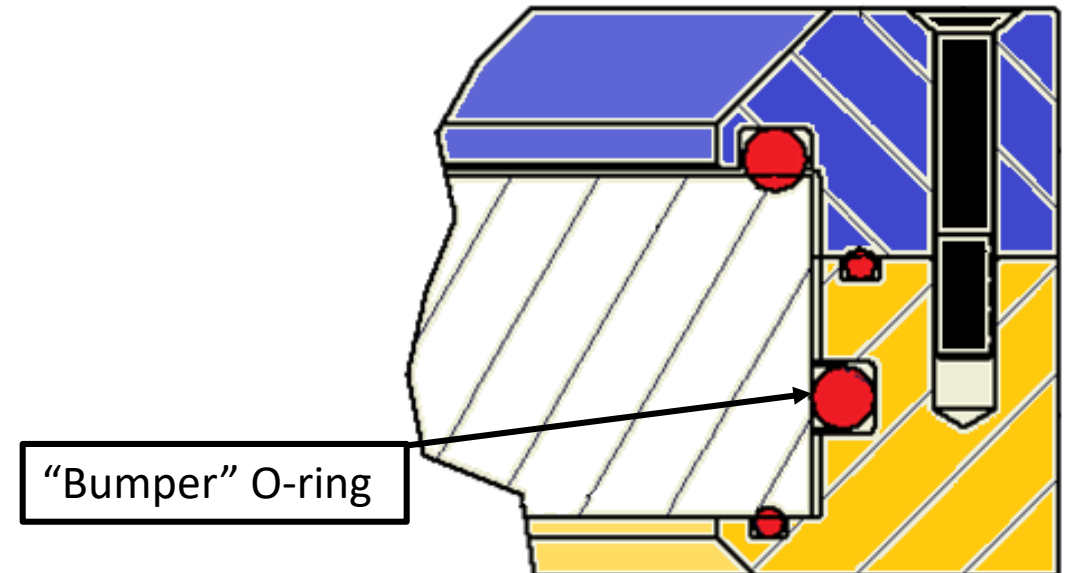
Glass Diameter	Viewport Diameter	Thickness	Material
16.875"	16.0"	1.125"	BK-7
5.90"	*5.0"	0.256"	BK-7
5.90"	*5.0"	0.256"	BK-7

*Later determined to be 4.6"

- Modify previously flown window casing system
 - "Bumper" O-ring hard to install
 - Previous flight this O-ring was not installed
 - Cause the edge of glass to crack



Window Casing System

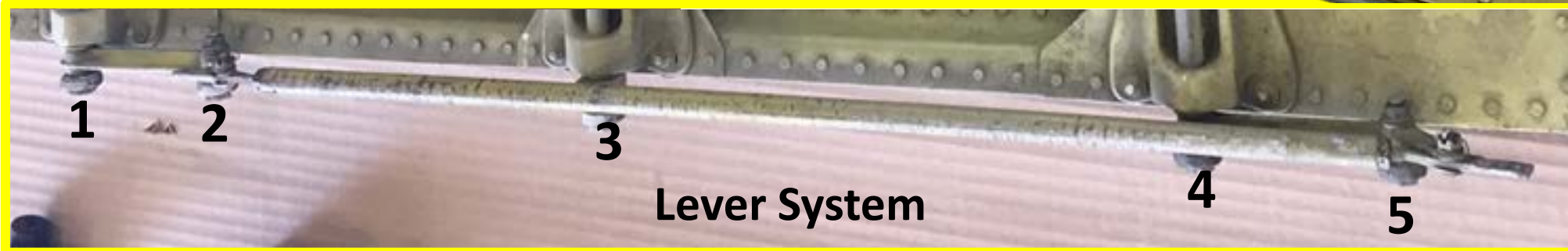
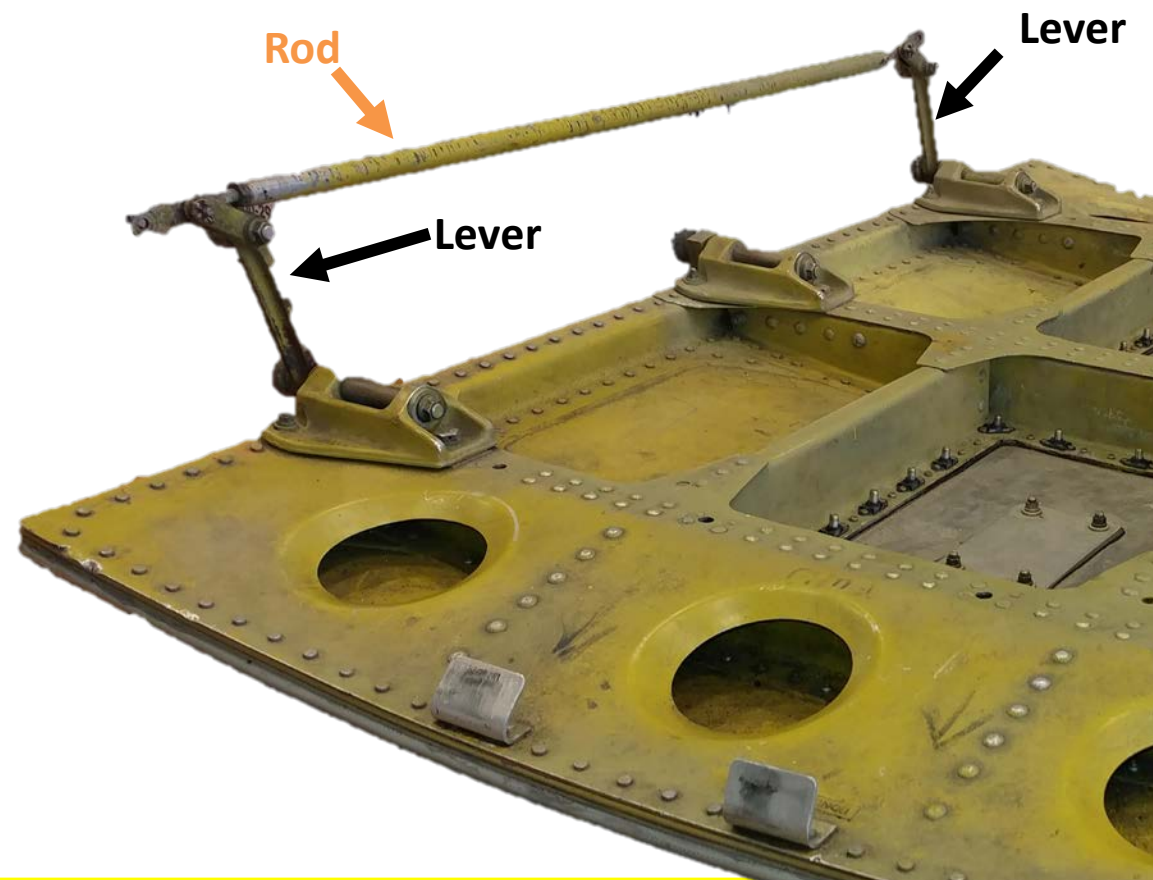


Cross section of Window Casing System

Introduction-Door

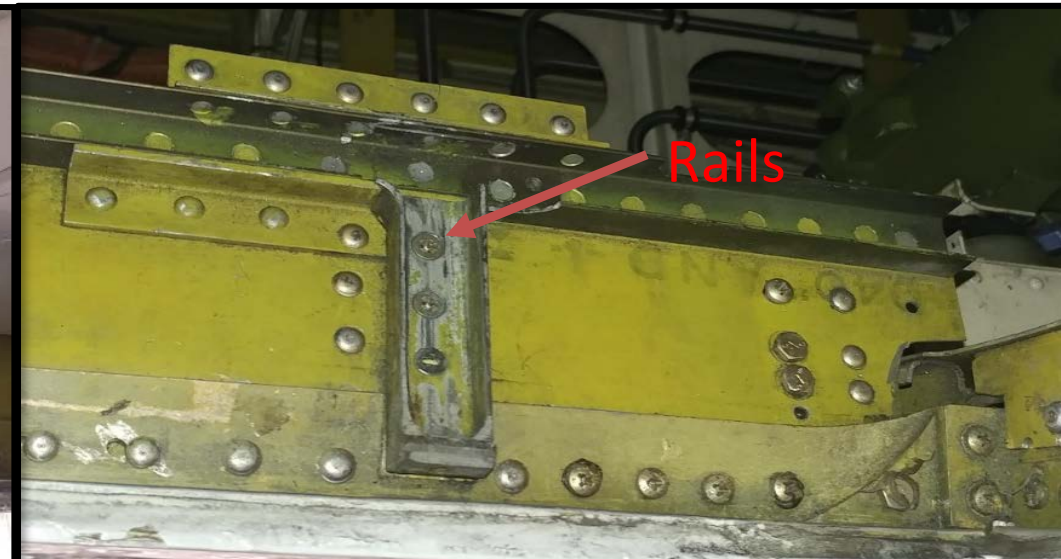
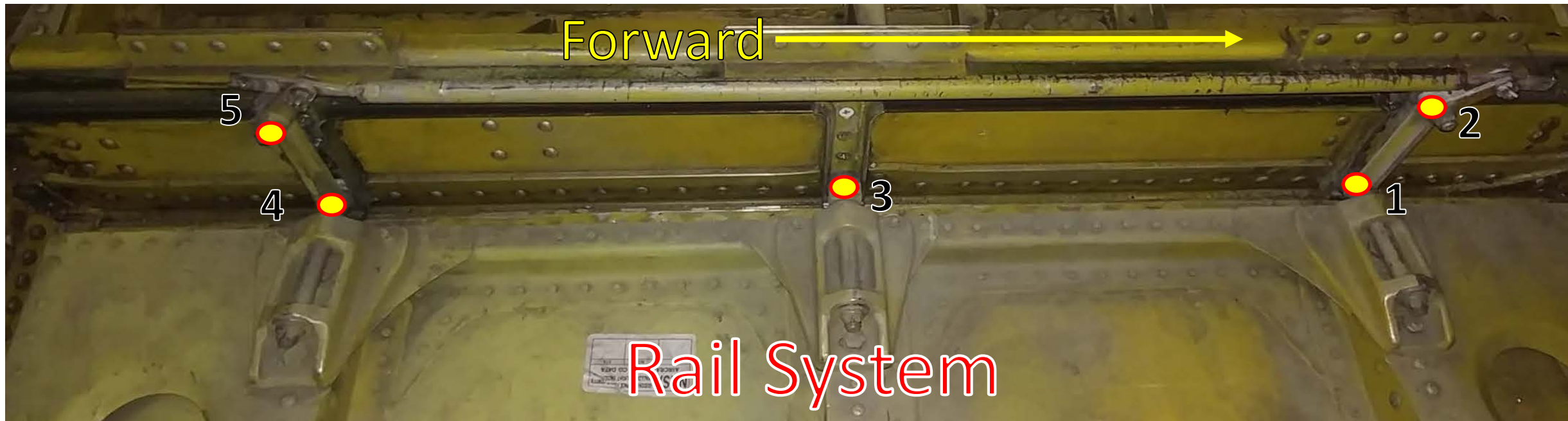


Seal



Lever System

Introduction-Aircraft Interface



Introduction-Historical Installs

Previous science installs use one of the modified original equipment doors shown.

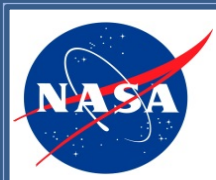
- Limited the science potential
 - Size and number of the viewports



Modified Door: 16" Circular viewport



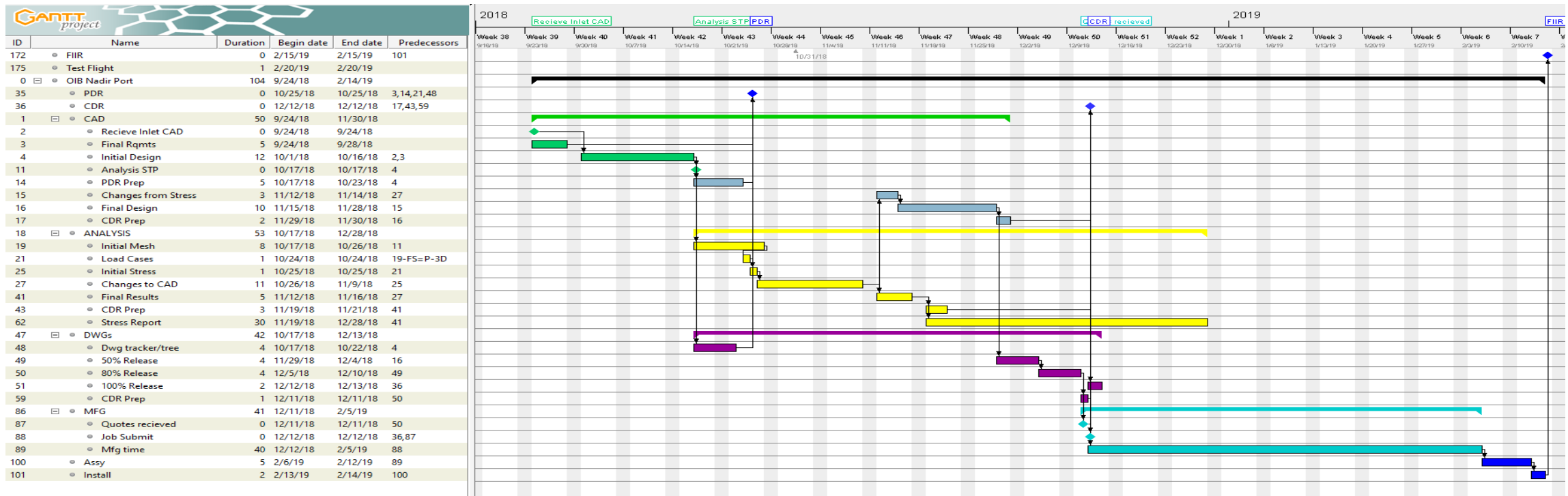
Modified Door: Rectangular viewports



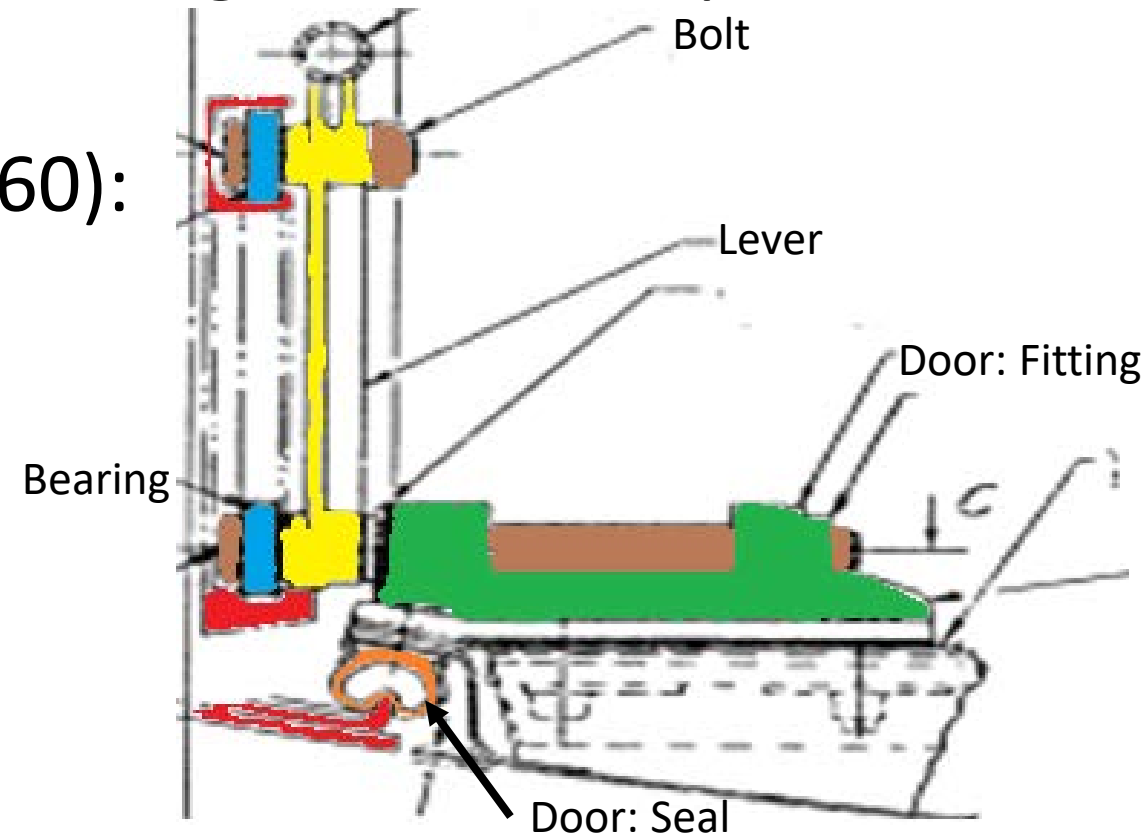
Design-Project Schedule



- Total time for design, analysis, drafting: ~4 Months
- Task assigned: ~ August 2018
- Parts to manufacturing: Dec. 10, 2018
- First mission: Jan. 2019



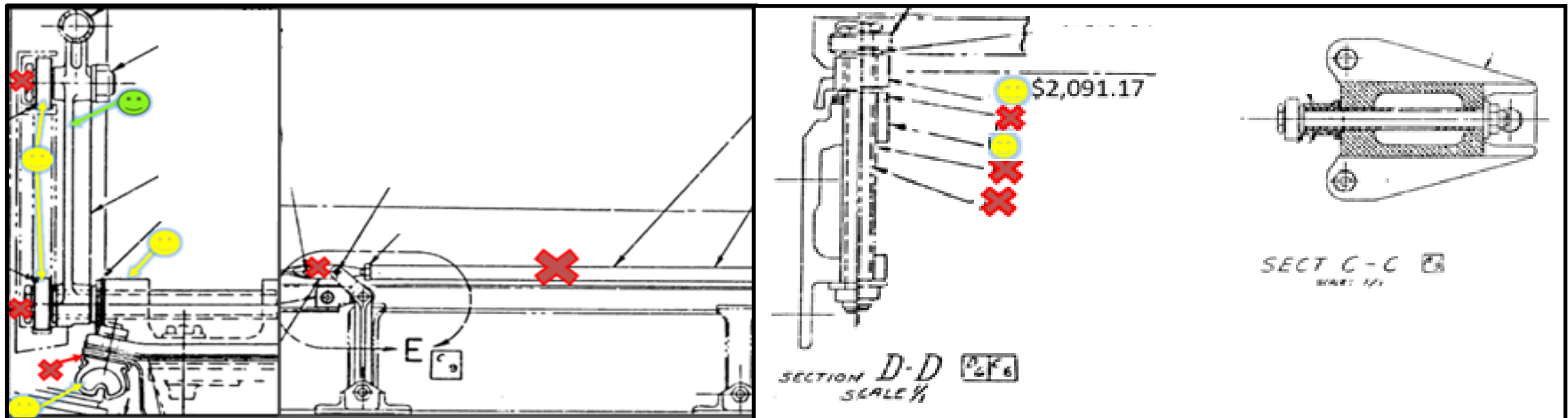
- Initial approach: Find records of the existing aircraft components and interface.
- LOCKHEED AIRCRAFT CORP (1959-1960):
 - **ASSY OF SONOBUOY DOOR**
 - **INSTALL OF SONOBUOY DOOR**
 - Lever Drawing (make-from drawing)



Cross Section view of
Door/Aircraft Interface

Design-Historical Records

-  In stock (ready to order)
-  In DLA (lead time unknown)
-  Not in DLA/discontinued



DLA: Defense Logistics Agency, government source to procure hardware

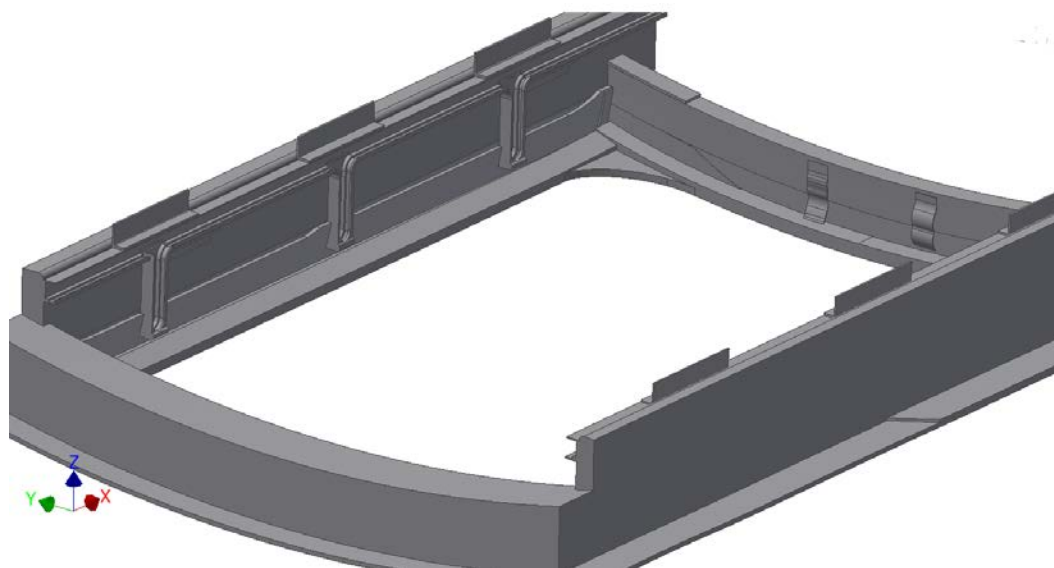
Design – Laser Scan: 3D Model

LASERDESIGN
A CyberOptics Corporation Brand

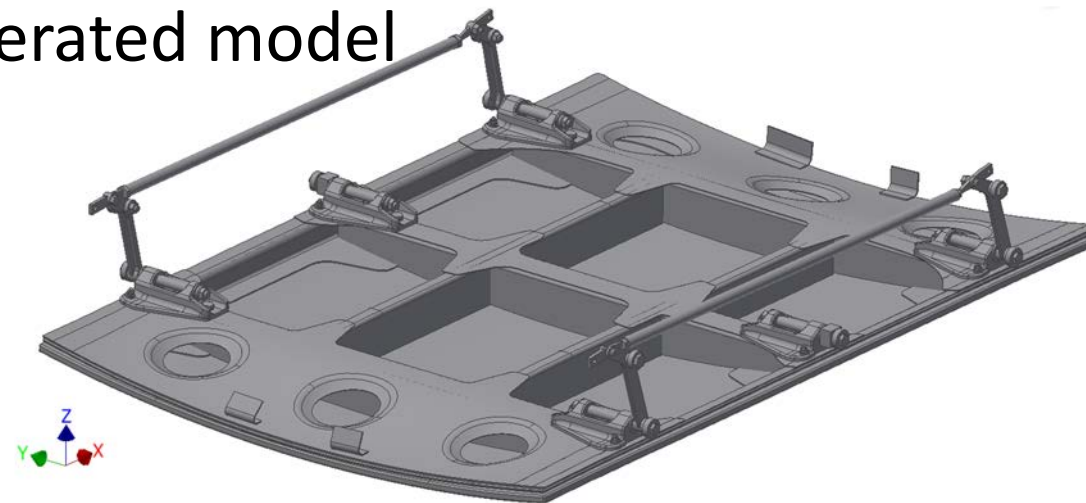
scanned the Nadir 1 and modified rectangular sonobuoy door.

- Generated a CAD from point cloud scan.
- Pictorial comparison of point cloud to generated model
- Received CAD Sept. 24, 2018

10 weeks remaining for reverse engineering

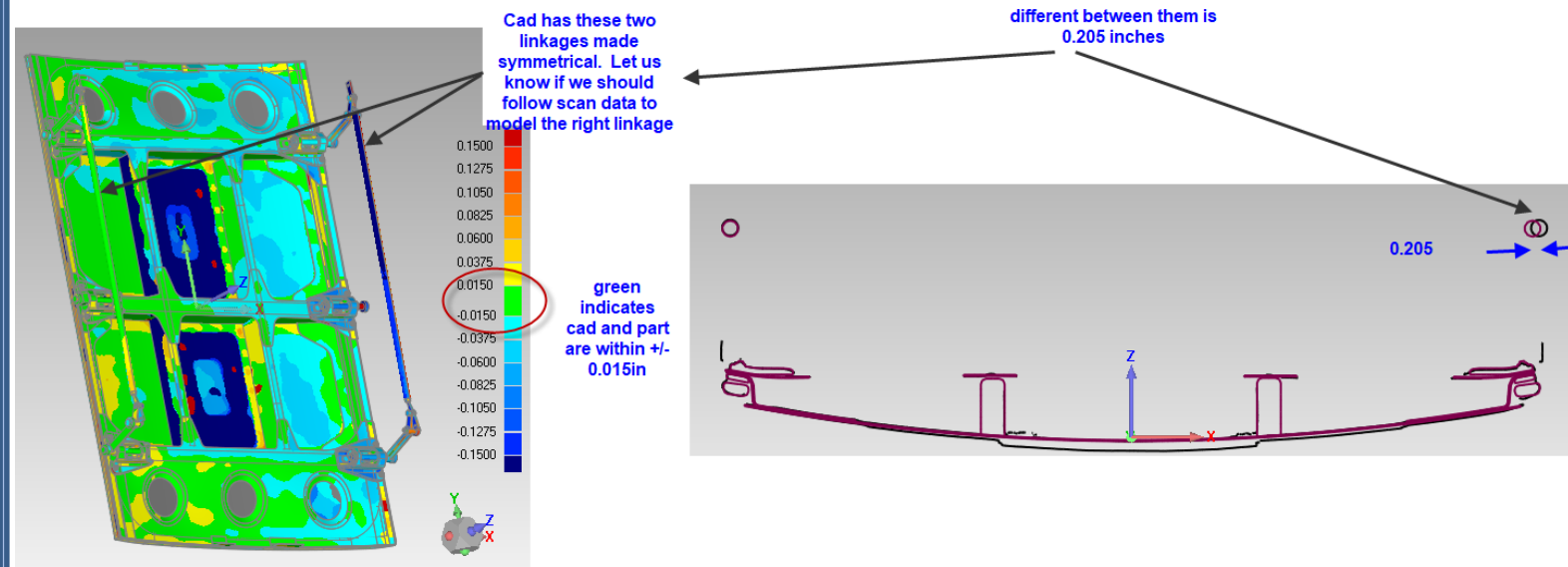


CAD Model of Aircraft



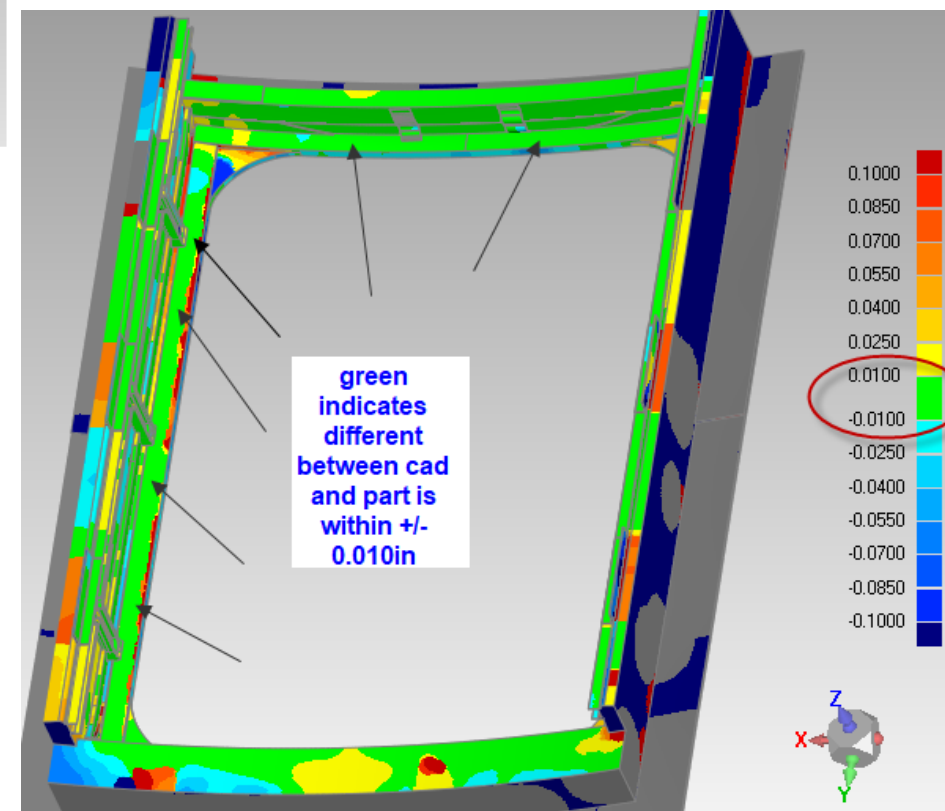
CAD Model of Door

Design – Laser Scan: 3D Model



Point Cloud and Door Model Comparison

- Comparison shows the door model within .015" and aircraft model within .010"



Point Cloud and Aircraft Model Comparison

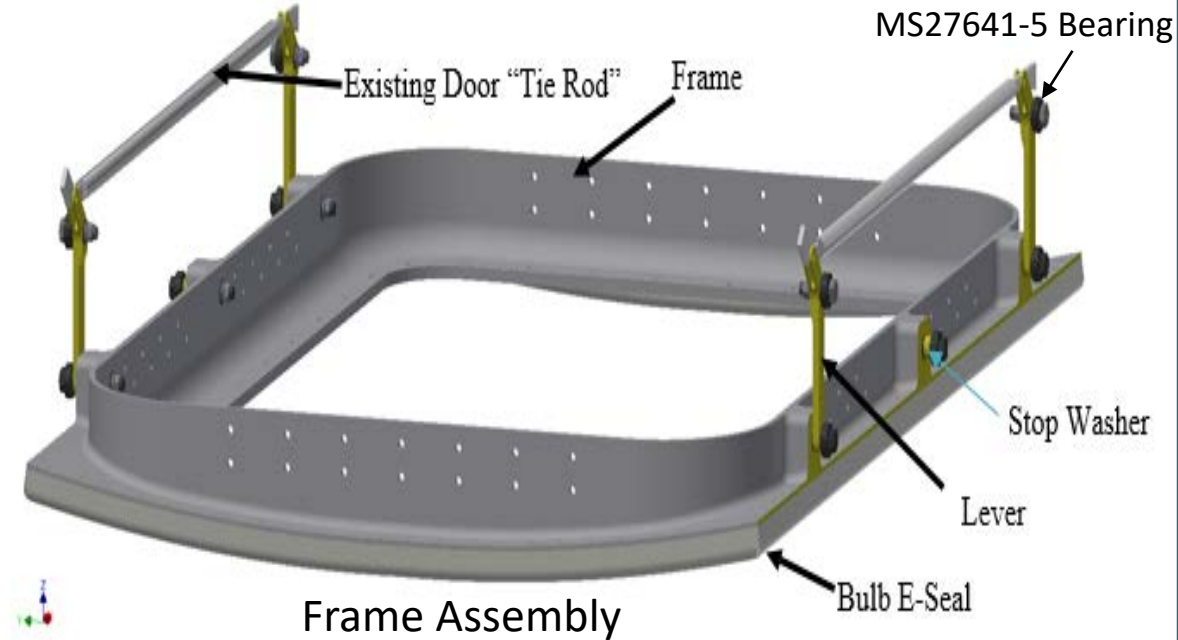
Final Design: Frame

- Milled 3.00" thick, Al 2024-T351 Plate (37" x 26")
- Science Opening: 20.24" x 31.05"
- Design allows for flat plate install

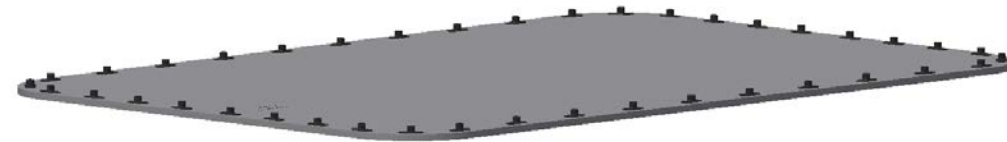


Frame

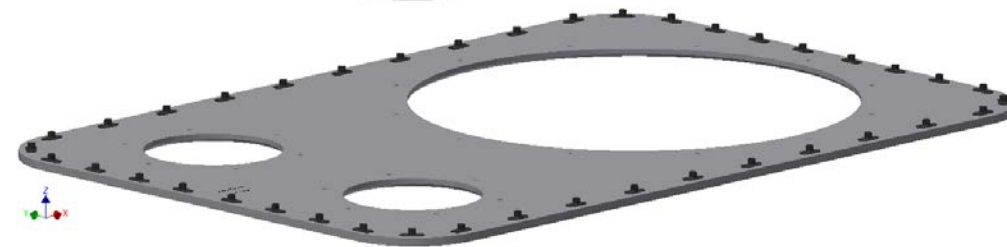
Weight: 14.0 lb (original 21 lb)



Frame Assembly

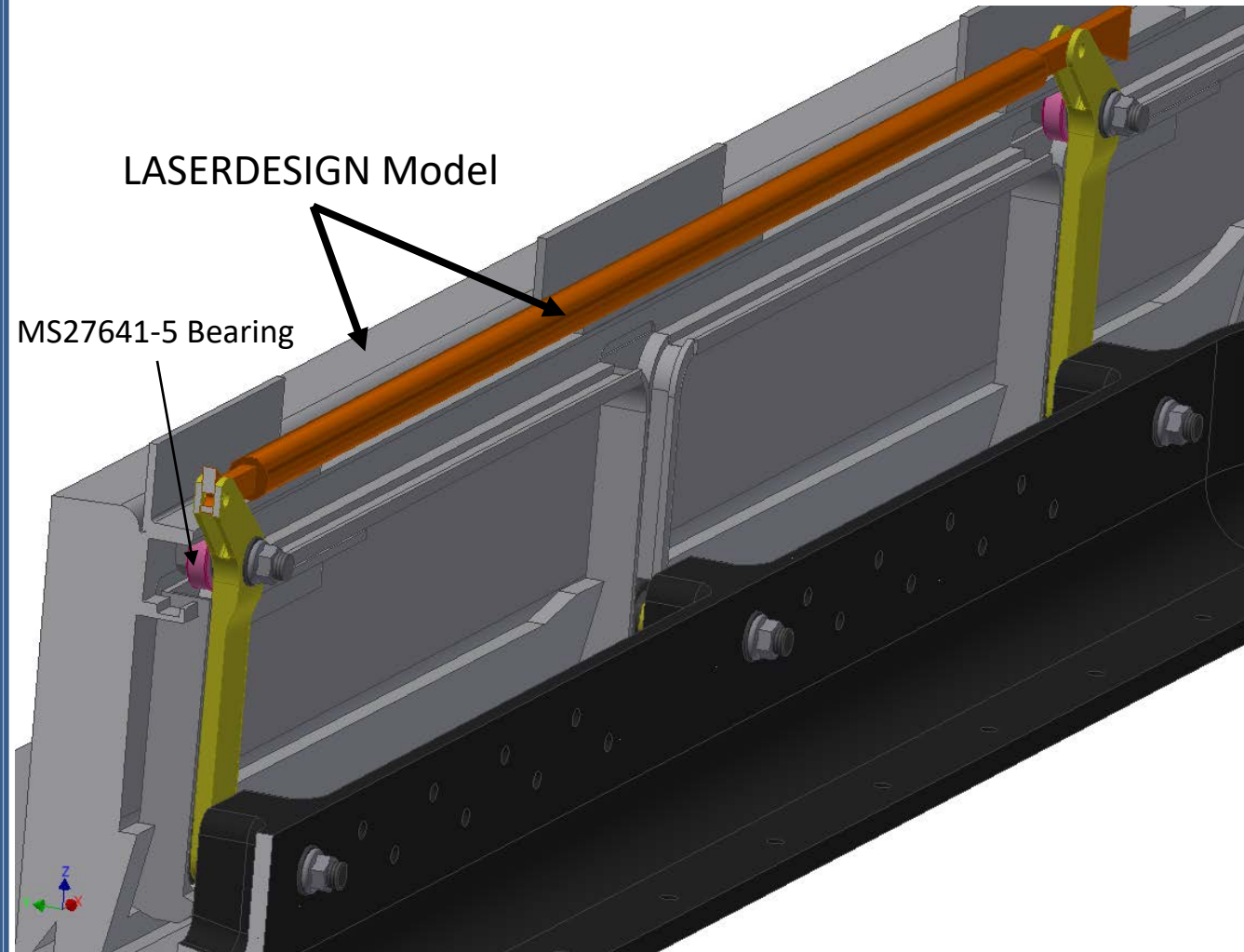


Cover Plate



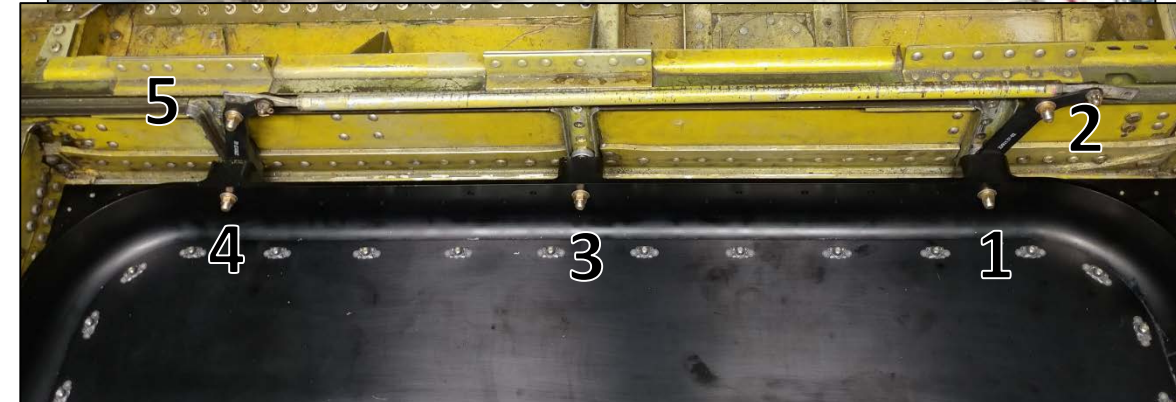
Science Plate

Final Design: Frame



CAD of Final Design with the LASERDESIGN Model

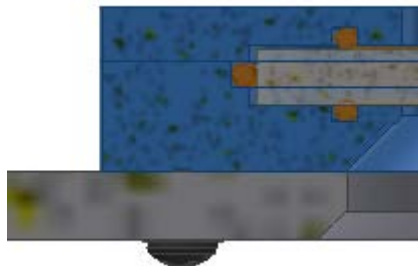
External view Final Design installed in aircraft



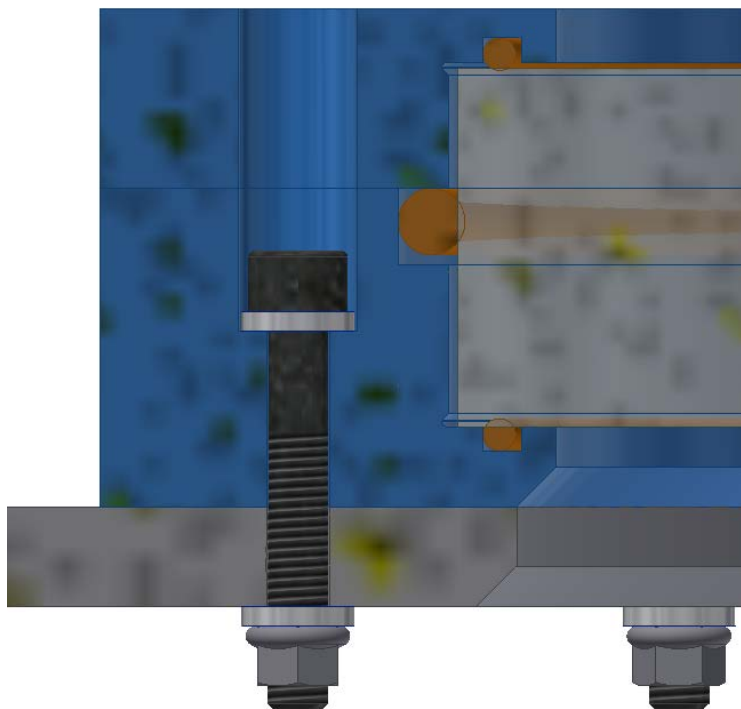
Internal view Final Design installed in aircraft

Design: Science

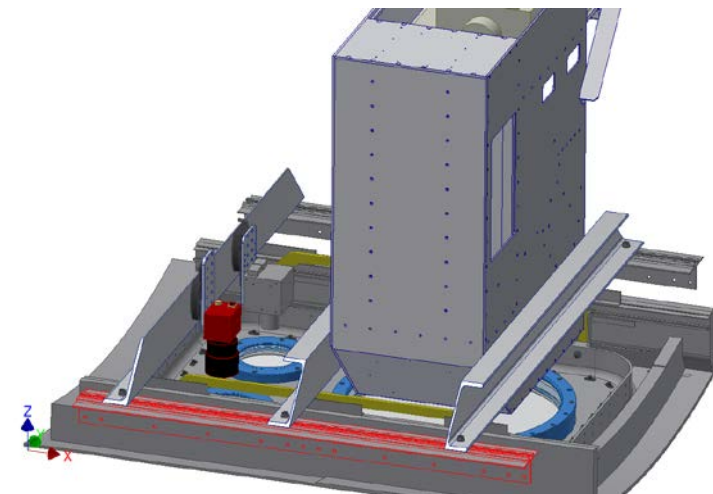
- 1x 16" Windows
- 2x 5" Windows



4.6" (5 ") Window

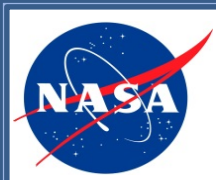


16" Window



← Forward





Load Cases



- Based on P-3B Requirements Document
 - 548-RQMT-001A
- Ultimate Pressure Loads (FS=2):
 - 11.98 psi
- Ultimate Inertial Loads (FS=2):
 - 3.0G Fwd
 - 10.2G Down
 - 6.4G Up
 - 3.2G Side
 - 1.5G Aft

Table 1. Minimum aircraft pressure vessel design criteria.*

Design Parameter	Pressure Limit (psi)
Maximum Cabin Differential Pressure	5.66
Maximum Emergency Relief Pressure (P)	5.99
Design Limit Pressure ($1.33P$)	7.97
Design Ultimate Pressure ($2P$)**	11.98

Table 4. Minimum emergency landing load criteria for the below cabin floor area.*

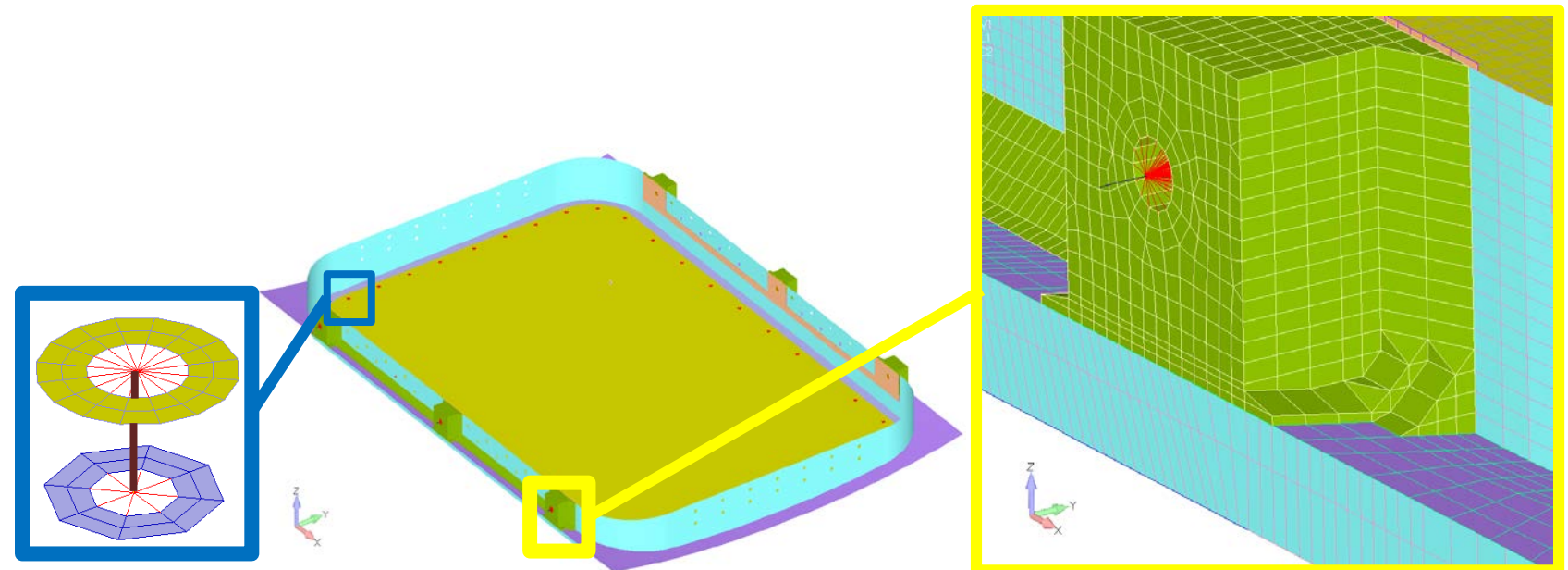
Load Direction	Ultimate Load Factor (g)
Forward	3.0
Down	4.5**
Up	2.0**
Lateral	1.5**
Aft	1.5

Table 8. Ultimate flight load factors FS280 to FS1130.²

Load Direction	Ultimate Load Factor (g)
Down	10.2
Up	6.4
Side	3.2

Frame Analysis Details

- Finite element model created in FEMAP version 11.2.2 and run through NX Nastran solver version 10.2
- Simplified the model to Plate (shell) elements & Solid at 6x Bolts that interact with the aircraft.
- Fasteners: bar elements with spidered rigid elements (RBE2) at the ends. RBE2's connect to the components on the nodes of the hole boundary.
- Non-structural mass is added to the flat plate property to simulate an additional equally distributed mass

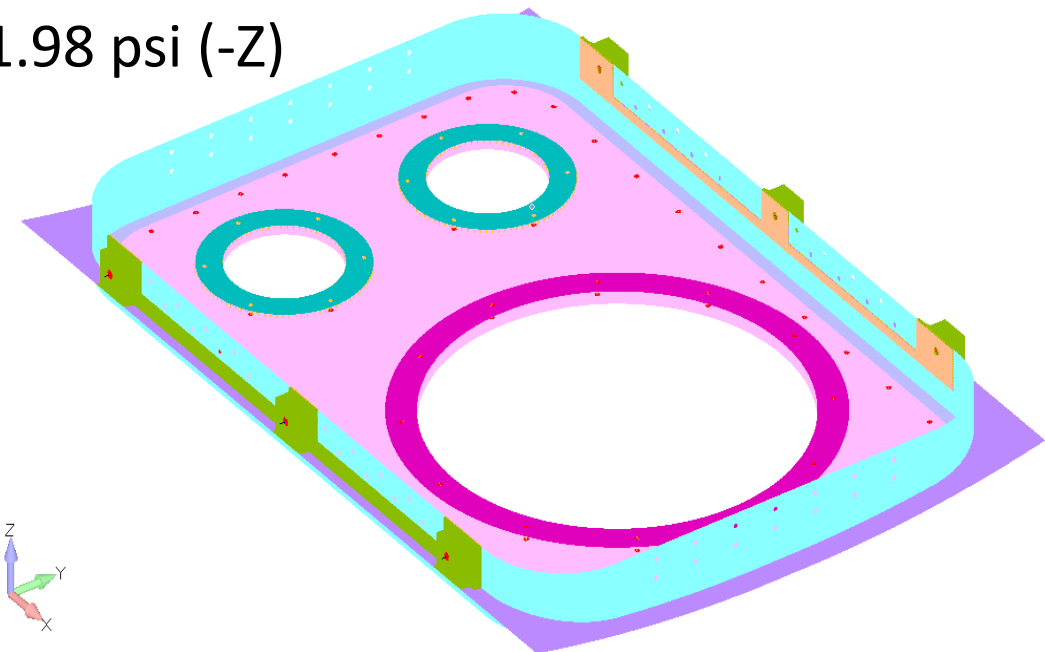
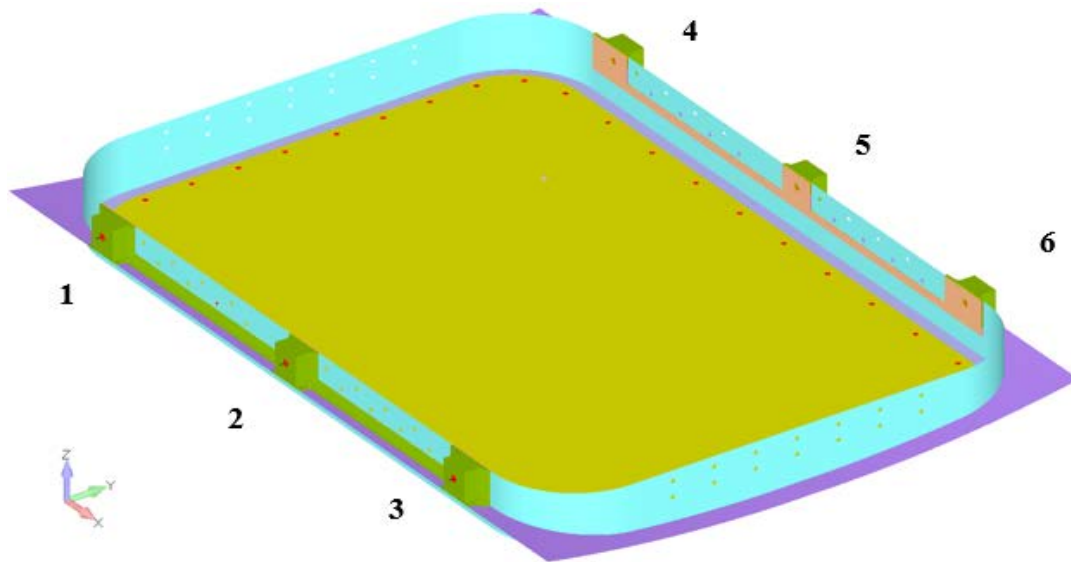


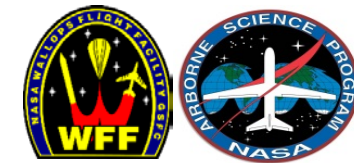
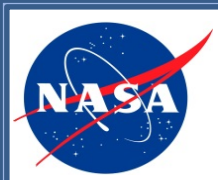
- Constraints:

Load Case	Bolt Constraint	Seal Constraint
10.2 G Down	1,2,3,4,5,6: Tx	Surface allows sliding (symmetry)
6.4 G Up	1,3,4,6: Tx, Ty, Tz; 2,5: Tx	NONE
3.0 G Forward	1,2,3,4,5,6: Tx	Surface allows sliding (symmetry)
3.2 G Side	4,5,6: Tx, Ty	Surface allows sliding (symmetry)
1.5 G Aft	1,2,3,4,5,6: Tx	Surface allows sliding (symmetry)

- Loads:

- Combined load case: Inertial Loads and 11.98 psi (-Z)

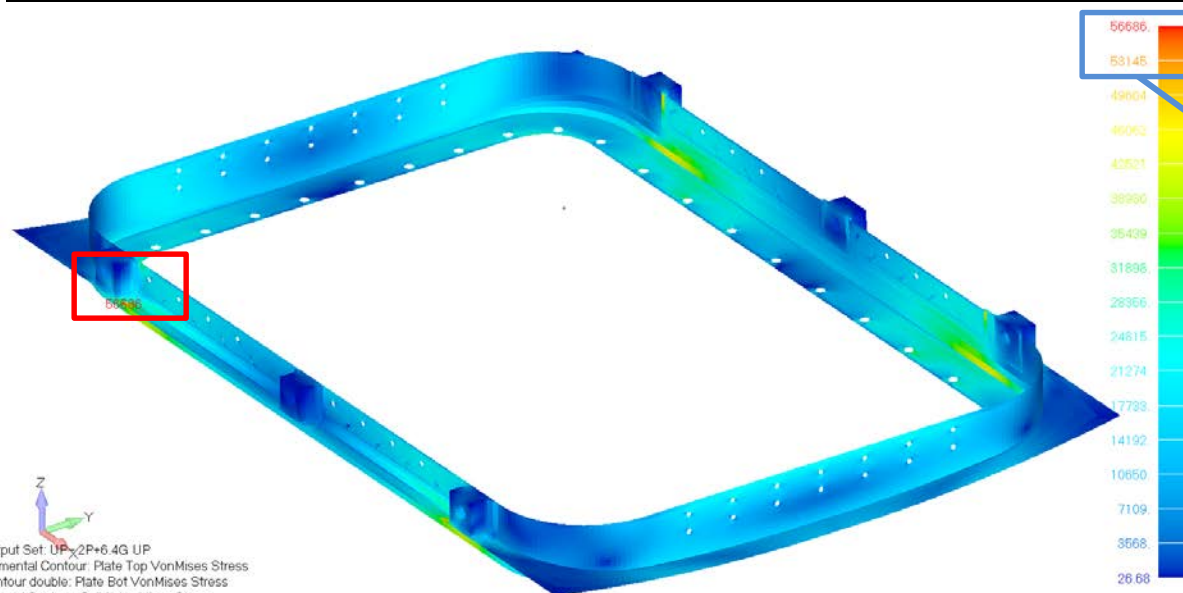




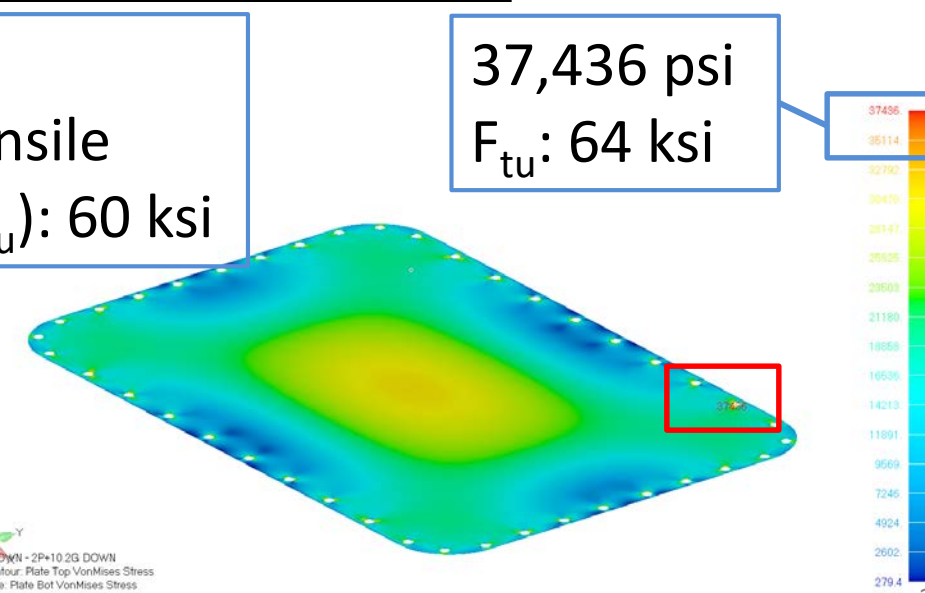
Analysis- Flat Plate Model

Summary of Results for Flat Plate Analysis Model

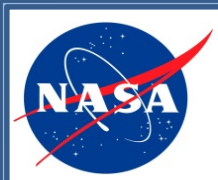
	Frame Von Mises Stress (psi)			Plate Von Mises Stress(psi)		
	17.5 lb Plate	59.2 lb Plate	115 lb Plate	17.5 lb Plate	59.2 lb Plate	115 lb Plate
10.2G Down + 2P	53,856	56,496	59,986	37,436	39,275	41,706
6.4G Up + 2P	56,686	54,871	52,471	35,903	34,759	33,247
Min MS	0.058	0.062	0.000	0.709	0.629	0.534
	Frame Shear Stress (psi)			Plate Shear Stress(psi)		
	17.5 lb Plate	59.2 lb Plate	115 lb Plate	17.5 lb Plate	59.2 lb Plate	115 lb Plate
10.2G Down + 2P	21,823	22,893	24,307	20,238	21,232	22,546
6.4G Up + 2P	27,600	26,717	25,549	19,065	18,457	17,654
Min MS	0.268	0.310	0.369	0.877	2.014	1.838



56,686 psi
Ultimate Tensile
Strength (F_{tu}): 60 ksi



37,436 psi
 F_{tu} : 64 ksi



Stress Results-Science Model

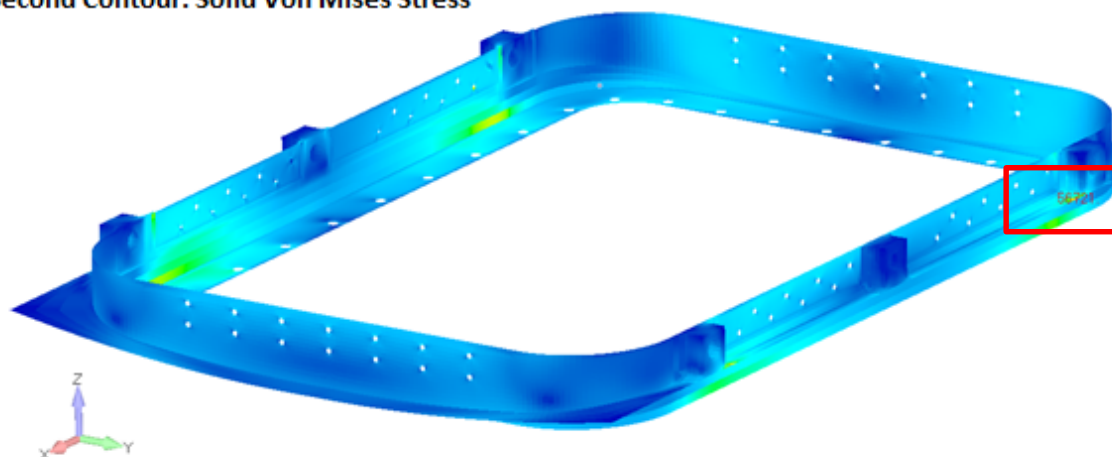
	Summary of Results for 59.2 lb Flat Plate Analysis Model	
	Frame Von Mises Stress	Plate Von Mises Stress
10.2G Down + 2P	56,496	39,275
6.4G Up + 2P	54,871	34,759
Min MS	.062	.629

Output Set: UP-2P+6.4G UP

Elemental Contour: Plate Top VonMises Stress

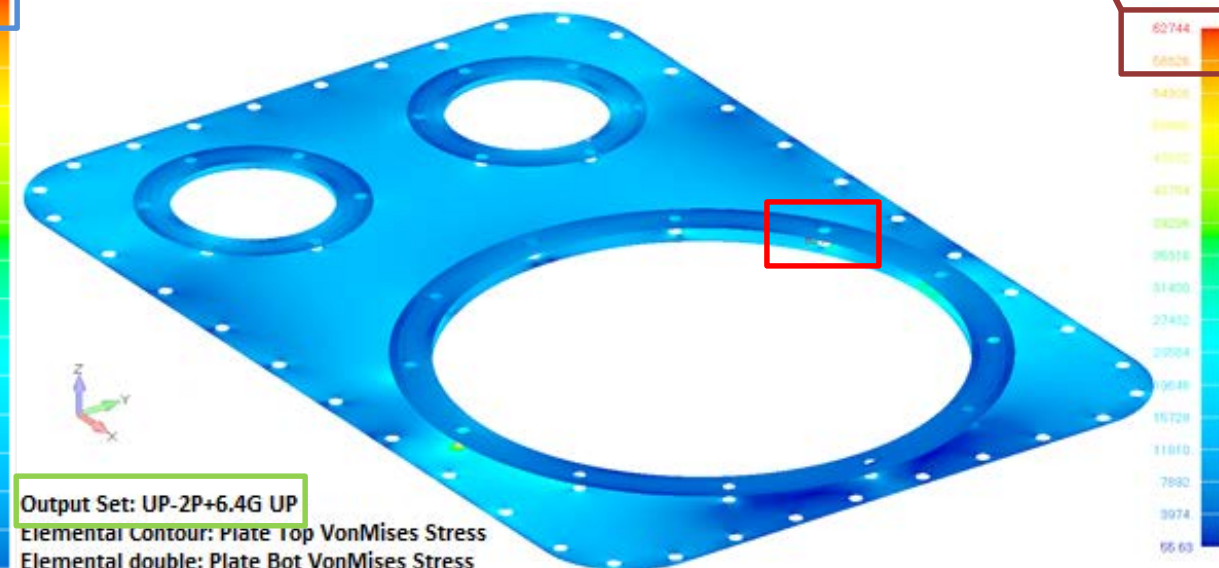
Contour double: Plate Bot VonMises Stress

Second Contour: Solid Von Mises Stress



56,721 psi
 F_{tu} : 60 ksi
MS=0.057

62,744 psi
 F_{tu} : 64 ksi
MS=0.020



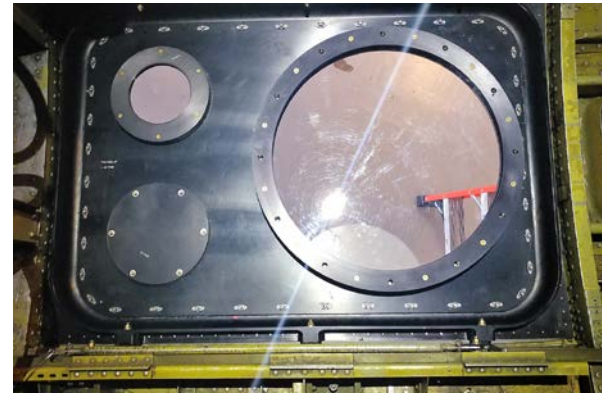
Output Set: UP-2P+6.4G UP

Elemental Contour: Plate Top VonMises Stress

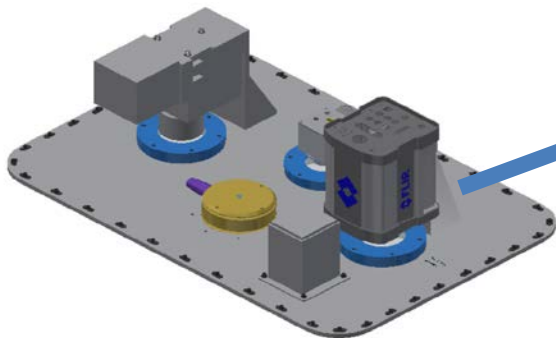
Elemental double: Plate Bot VonMises Stress

Conclusion

- Designed a fixture that increased the capability of the WFF P-3B Aircraft
- Science can be outfitted on a **flat** plate with a 20.24" x 31.05" opening
- The scanning technology open the potential to further improve aircraft science capabilities

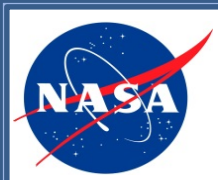


OIB Plate Install



CAMP²Ex Plate Install

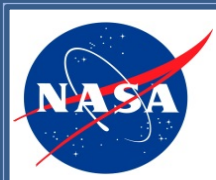




Lessons Learned & Challenges

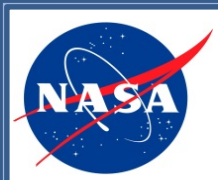


- Challenges:
 - Constrained schedule
 - Plan for shutdown
 - Lack of historical references & reverse engineer from these references
 - Request 3D scans in August, and use of scan was critical to design
 - Communicating the use of 3D scan technology with senior members
- Lessons learned
 - Don't depend on only one source (Historical References)
 - Don't let schedule impact engineering



Thank you

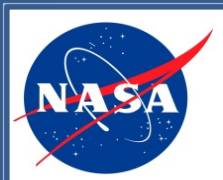
Questions?



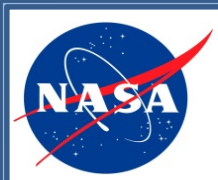
Acronym List



AC3	Axial Cyclone Cloud-Water Collector
CAD	Computer Aided Design
CAMP ² Ex	Cloud, Aerosol and Monsoon Processes Philippines Experiment
FEMAP	Finite Element Modeling and Postprocessing
F_{tu}	Ultimate Tensile Strength
GSFC	Goddard Space Flight Center
IRAD	Internal Research and Development
OIB	Operation Ice Bridge
WCM	Water Content Multi-element System
WFF	Wallops Flight Facility



BACK UP SLIDES



Model Verification

The inertial loads are applied as body loads. The pressure load is applied as elemental pressure to the plate, whose surface area is 712 in². As a modeling check, resultant loads are extracted from the .F06 file and show that they are equal but opposite to the applied loads.

		1 G Weight (lbs)													
Frame		14.0													
Plate		17.5													
Total 1G Weight		31.5													

	10.2G Dwn (lbs)			6.4G Up (lbs)			3.0G Fwd (lbs)			3.2G Side (lbs)			1.5G Aft (lbs)		
	x	y	z	x	y	z	x	y	z	x	y	z	x	y	z
Pressure	0	0	-8546	0	0	-8546	0	0	-8546	0	0	-8546	0	0	-8546
Inertial	0	0	-322	0	0	202	-95	0	0	0	101	0	47	0	0
Total	0	0	-8869	0	0	-8344	-95	0	-8546	0	101	-8546	47	0	-8546

SPCFORCE RESULTANT			
LOAD CASE	FX	FY	FZ
10.2 G Down	0	0	8,869
6.4G Up	0	0	8,344
3.0G Fwd	95	0	8,546
3.2G Side	0	-101	8,546
1.5G AFT	-47	0	8,546

Analysis Results

- Additional analysis included:

- Buckling analysis of the Lever

The buckling load is calculated for the 6.4G UP load without the 2P pressure.

$$F_{17.5(buckling)} = 6.4 * (17.5 + 14) = 205 \text{ lbs}$$

$$F_{115(buckling)} = 6.4 * (115 + 14) = 826 \text{ lbs}$$

Output Set: Eigenvalue 1 18.9394
Deformed(1.): Total Translation



Output Set: Eigenvalue 1 4.700456
Deformed(1.): Total Translation



- Lever strength analysis

$$P_{u(tension)} = F_{tu} * (2R - D) * t$$

$$= 63 * 10^3 (.687 - .328) * .2 = 4,523 \text{ lbs}$$

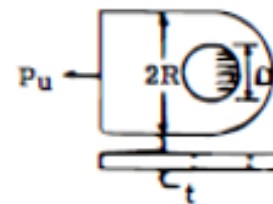
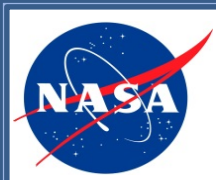


Fig. D1.8

Bolt and Lug Strength Analysis, Bruhn's Fig. D1.8



Analysis Results-Continued



- Bolt analysis using load extracted from the bar elements in the finite element model

7.2.5 #10-32 Bolt Analysis

The #10-32 bolts are used to attach the plate to the frame. Bar forces are extracted from the FEM model and used to conduct the analysis. Table 10 shows the critical loads for the bolts for the 10.2G Down and 6.4G Up load cases for the 17.5 and 115 lb plates. The worst case adjusted loads (Load*FF) occur for the 17.5 lb plate and are 686 lb. in shear and 509 lb. in tension. These values are used in the MS_{shear} , $MS_{tension}$, and interaction equation.

$$MS_{shear} = \frac{2,690}{686} - 1 = 2.92$$

$$MS_{tension} = \frac{2,890}{509} - 1 = 4.68$$

$$\left(\frac{(MS + 1) * 686}{2,690} \right)^2 + \left(\frac{(MS + 1) * 509}{2,890} \right)^2 = 1 \xrightarrow{\text{yields}} MS = 2.23$$

The max shear load (686 lb.) is used to conduct a shear tear-out and bearing stress calculation where $D = 0.19$, and $t = 0.25$ (thickness of plate). The frame thickness is 0.30" but has lower shear and bearing strength, 35 ksi and 91 ksi respectively. The MS shear and bearing are 7.72 and 6.56 respectively.

$$\sigma_{shear} = \frac{686 \text{ lb}}{0.1425 \text{ in}^2} = 4,812 \text{ psi}$$

$$MS_{shear} = \frac{\sigma_{allowable}}{Max \sigma} - 1 = \frac{38 \text{ ksi}}{4,812 \text{ psi}} - 1 = 6.90$$

$$\sigma_{bearing} = \frac{686 \text{ lb}}{0.0475 \text{ in}^2} = 14,436 \text{ psi}$$

$$MS_{bearing} = \frac{\sigma_{allowable}}{Max \sigma} - 1 = \frac{97 \text{ ksi}}{14,436 \text{ psi}} - 1 = 5.72$$

7.2.6 5/16-24 Bolt Analysis

$$F_{prying} = \frac{\sqrt{M_{plane1}^2 + M_{plane2}^2}}{0.66 * e} = \frac{\sqrt{(-523)^2 + 451^2}}{0.66 * 0.406} = 2,544$$

$$F_{tension} = 3,350 + 1.15 * (2,544) = 6,276$$

$$MS_{shear} = \frac{7,290}{3,168} - 1 = 1.30$$

$$MS_{tension} = \frac{8,590}{6,276} - 1 = 0.37$$

$$\left(\frac{(MS + 1) * 3,168}{7,290} \right)^2 + \left(\frac{(MS + 1) * 6,276}{8,590} \right)^2 = 1 \xrightarrow{\text{yields}} MS = 0.18$$

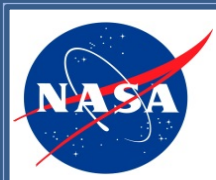
The max shear load (3,168 lb.) is used to conduct a shear tear-out and bearing stress calculation where $D = 0.3125$, and $t = 0.2$ (thickness of lever). The frame thickness is 1.03" but has lower shear and bearing strength, 35 ksi and 91 ksi respectively. The MS shear and bearing are 9.67 and 8.24 respectively.

$$\sigma_{shear} = \frac{3,168 \text{ lb}}{0.1875 \text{ in}^2} = 16,894 \text{ psi}$$

$$MS_{shear} = \frac{\sigma_{allowable}}{Max \sigma} - 1 = \frac{37 \text{ ksi}}{16,894 \text{ psi}} - 1 = 1.190$$

$$\sigma_{bearing} = \frac{3,168 \text{ lb}}{0.0625 \text{ in}^2} = 50,682 \text{ psi}$$

$$MS_{bearing} = \frac{\sigma_{allowable}}{Max \sigma} - 1 = \frac{95 \text{ ksi}}{50,682 \text{ psi}} - 1 = 0.874$$



Analysis Results-Continued



- Glass window Analysis

8.2 Results Summary

This section contains the calculation results from the window Analysis. Each window of the installation is looked at separately for peak stress values.

Table 14: Stress Summary

Component	Combined Stress (psi)	MS	Flaw Allowance Stress (psi)	MS	Min MS
16" Window	1,749	2.20	5,247	0.07	0.07
5" Window	1,838	2.05	5,513	0.01	0.01

8.2.1 16" Window Analysis

$$\Delta T = \left| -2.65 - \frac{\frac{0.64(-2.65)}{(1.125/12)} + 10 \times 70}{\frac{0.64}{(1.125/12)} + 10} \right| = 43.18$$

$$S_{max} = \frac{3(5.99)(3 + 0.207)8.112^2}{8(1.125^2)} = 375 \text{ psi} \times 2 = 749 \text{ psi}$$

$$\sigma_{th} = \frac{(3.9 \times 10^{-6})(1.18 \times 10^7)43.18}{4} = 497 \text{ psi} \times 1.2 = 596 \text{ psi}$$

$$\sigma_{st} = 749 + 596 = 1,345 \text{ psi} \times 1.3 = 1,749 \text{ psi}$$

$$MS = \frac{5,600 \text{ psi}}{1,749 \text{ psi}} - 1 = 2.20$$

$$\sigma_{flaw} = 3 \times 1,749 = 5,247 \text{ psi}$$

$$MS = \frac{5,600 \text{ psi}}{5,247 \text{ psi}} - 1 = 0.07$$

8.2.2 5" Window Analysis

$$\Delta T = \left| -2.65 - \frac{\frac{0.64(-2.65)}{(0.256/12)} + 10 \times 70}{\frac{0.64}{(0.256/12)} + 10} \right| = 18.16$$

$$S_{max} = \frac{3(5.99)(3 + 0.207)2.3^2}{8(0.256^2)} = 581 \text{ psi} \times 2 = 1,163 \text{ psi}$$

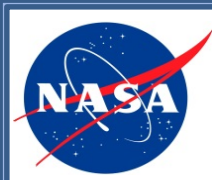
$$\sigma_{th} = \frac{(3.9 \times 10^{-6})(1.18 \times 10^7)18.16}{4} = 209 \text{ psi} \times 1.2 = 251 \text{ psi}$$

$$\sigma_{st} = 1,163 + 251 = 1,414 \text{ psi} \times 1.3 = 1,838 \text{ psi}$$

$$MS = \frac{5,600 \text{ psi}}{1,838 \text{ psi}} - 1 = 2.05$$

$$\sigma_{flaw} = 3 \times 1,838 = 5,513 \text{ psi}$$

$$MS = \frac{5,600 \text{ psi}}{5,513 \text{ psi}} - 1 = 0.0157$$



MMPDS- Data Sheet



Table 3.2.4.0(b₂). Design Mechanical and Physical Properties of 2024 Aluminum Alloy Sheet and Plate

Specification	AMS 4037*											
Form	Plate											
Temper	T351											
Thickness, in.	0.250-0.499		0.500-1.000		1.001-1.500		1.501-2.000		2.001-3.000		3.001-4.000	
Basis	A	B	A	B	A	B	A	B	A	B	A	B
Mechanical Properties:												
F_u , ksi:												
L	64	66	63	65	62	64	62	64	60	62	57	59
LT	64	66	63	65	62	64	62	64	60 ^d	62 ^d	57 ^d	59 ^d
ST	...	...	...	...	...	...	...	...	52 ^b	54 ^b	49 ^b	51 ^b
F_u , ksi:												
L	48	50	48	50	47	50	47	49	46	48	43	46
LT	42	44	42	44	42	44	42	44	42	44	41	43
ST	...	...	...	...	...	...	...	...	38 ^b	40 ^b	38 ^b	39 ^b
F_{cy} , ksi:												
L	39	41	39	41	39	40	38	40	37	39	35	37
LT	45	47	45	47	44	46	44	46	43	45	41	43
ST	...	...	...	...	...	...	...	...	46	48	44	47
F_{uT} , ksi (L & LT)	38	39	37	38	37	38	37	38	35	37	34	35
F_{brs} , ksi:												
L & LT (e/D = 1.5)	97	100	95	98	94	97	94	97	91	94	86	89
L & LT (e/D = 2.0)	119	122	117	120	115	119	115	119	111	115	106	109
F_{brs} , ksi:												
L & LT (e/D = 1.5)	72	76	72	76	72	76	72	76	72	76	70	74
L & LT (e/D = 2.0)	86	90	86	90	86	90	86	90	86	90	84	88
σ , percent (S-Basis):												
LT	12	...	8	...	7	...	6	...	4	...	4	...
E , 10 ³ ksi	10.7											
E_c , 10 ³ ksi	10.9											
G , 10 ³ ksi	4.0											
μ	0.33											
Physical Properties:												
ω , lb/in ³	0.100											
C, K	See Figure 3.2.4.0(a)											
α	See Figure 3.2.4.0(b)											

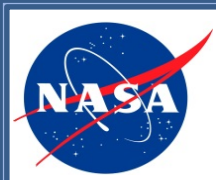
Last Revised: Apr 2015, MMPDS-10, Item 14-35. Design allowables were last confirmed in Item 07-41, MMPDS-04CN1.

a Mechanical properties were established under QQ-A-250/4.

b Caution: This specific alloy, temper, and product form exhibits poor stress corrosion cracking resistance in this grain direction. It corresponds to an SCC resistance rating of D, as indicated in Table 3.1.2.3.1(a).

c Bearing values are "dry pin" values per Section 1.4.7.1. See Table 3.1.2.1.1.

d The following rounded T_{90} and T_{99} values represent production capacity at the time the table was last confirmed; F_u LT for 2-3 inches T_{90} = 63 ksi, T_{99} = 64 ksi; for 3-4 inches T_{90} = 60 ksi, T_{99} = 62 ksi.



Material Certifications- Frame

AMAG
ROLLING

Abnahmeprüfzeugnis 3.1 (EN 10204)
Inspection certificate - mill certificate

Nr.: 89099446 01 / 1
Rev.: 0
Seite / page: 1 von / of 2
Datum / date: 2018 07 09

Zertifiziert nach / certified to ISO 9001, ISO/TS 16949, ENAS 9100, ISO 14001, NADCAP

Auftraggeber / customer:
Alro Steel Corporation
3100 E. High Street
JACKSON MI 49204-0927
USA

Warenempfänger / consignee:
Alro Steel Corporation
5859 Alro Park Drive
POTTERVILLE MI 48876
USA

Bestell Nr. / order no.: 13379301
Datum / date: 2018 04 10

Auftragsbest. Nr. / order confirm no.: 147180
Datum / date: 2018 04 11

Lieferschein Nr. / delivery note: 89099446
Datum / date: 2018 07 31

Produkt / product
Form / form: Plate, stretched
Werkstoff / material: 2024
Zustand / temper: T351
Dim. / dim.: [inch]: 3,000x48,50x144,50
Kundenartikel Nr. / cust. article no.: 27922500

Bedingungen / terms
Technische Lieferbedingungen / techn. spec.:
AMS-QQ-A-250/4 Rev.B, 2015-02
ASTM B 209 - 14
AMS 4037Q, 12 2014
Sondervorschrift / special terms:
not US tested

AB-Pos. ord. item	BNr/Los Lot/No.	Teillos Part	Guss Nr. cast no.	Werkstoff material	Kollo- packno.	Gewicht netto weight net	Stk. pcs.
04	76288/01	00	01/0092766/8	2024	1471800011	4307,760 lbs	2
04	76288/01	00	01/0092766/8	2024	1471800012	4312,169 lbs	2

Chemische Zusammensetzung [%] (Gewichtsanteile) / Chemical composition [%] weight proportion (GES)

Guss Nr. / cast no.	material	Si Fe Cu Mn Mg Cr Zn Ti Others Each Others Total																	
		spec. min.	spec. max.	actual	spec. min.	spec. max.	actual	spec. min.	spec. max.	actual	spec. min.	spec. max.	actual	spec. min.	spec. max.	actual	spec. min.	spec. max.	actual
01/0092766/8	2024	-	-	3,8	0,30	1,2	-	-	-	-	-	-	-	-	-	-	-	-	-
		0,50	0,50	4,9	0,9	1,8	0,10	0,25	0,15	0,05	0,15								
		0,05	0,09	4,3	0,57	1,5	0,03	0,07	0,04	0,02	0,03								

Zugprüfung LT / tensile test LT

BNr/Los Lot/No.	Zustand temper	Richtung direction	Tests	UTS [ksi]	YS [ksi]	A2" [%]	
			spec. min.	60,0	42,0	4	
			spec. max.	-	-	-	
76288/01	T351	LT	3	from	67,7	47,0	16
76288/01	T351	LT		to	68,0	47,8	17

Sonstige Prüfungen / other tests
Maßkontrolle: OK / Dimensional Check: OK
Oberfläche: OK / Surface inspection: OK

RECEIVED
AUG 17 2018
ALRO STEEL PV

Produkt / product
Form / form: Plate, stretched
Werkstoff / material: 2024
Zustand / temper: T351
Dim. / dim.: [inch]: 3,000x48,50x144,50
Kundenartikel Nr. / cust. article no.: 27922500

Bedingungen / terms
Technische Lieferbedingungen / techn. spec.:
AMS-QQ-A-250/4 Rev.B, 2015-02
ASTM B 209 - 14
AMS 4037Q, 12 2014
Sondervorschrift / special terms:
not US tested

AB-Pos.	BNr/Los	Teillos	Guss Nr.	Werkstoff	Kollo	Gewicht netto	Stk.
04	76288/01	00	01/0092766/8	2024	1471800011	4307,760 lbs	2
04	76288/01	00	01/0092766/8	2024	1471800012	4312,169 lbs	2

BNr/Los Lot/No.	Zustand temper	Richtung direction	Tests	UTS [ksi]	YS [ksi]	A2" [%]	
			spec. min.	60,0	42,0	4	
			spec. max.	-	-	-	
76288/01	T351	LT	3	from	67,7	47,0	16
76288/01	T351	LT		to	68,0	47,8	17

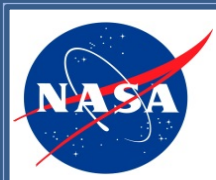
Sonstige Prüfungen / other tests

Alro Metals/Plastics



RT09406262

AMAG rolling GmbH, Postfach 32 A-5282 Ranshofen, Österreich - www.amag.at
AMAG rolling GmbH, P.O. Box 32, A-5282 Ranshofen, Austria - www.amag.at



Material Certifications- Plate



SHIP TO: AFFILIATED - DIV OF BRALCO 450 N BILLY MITCHELL ROAD SALT LAKE CITY, UT 84116						KAISER ALUMINUM Trentwood Works - Spokane, WA 99215 Phone: (800) 367-2586 CERTIFIED TEST REPORT Serial Number 4473869							
SOLD TO: AFFILIATED - DIV OF BRALCO 450 N BILLY MITCHELL ROAD SALT LAKE CITY, UT 84116													
CUSTOMER PO NUMBER: 16-10514-1		WORK PACKAGE:		CUSTOMER PART NUMBER:		SHIP RUN/LOAD: 104009/9		GOV'T CONTRACT NUMBER:					
KAISER ORDER NO: 1239605-1		SHIP DATE: 13-AUG-2017		ALLOY: 2024		CLAD: BARE		TEMPER: T351		PRODUCT DESCRIPTION: Sawed Plate			
WEIGHT SHIPPED: 4820 LB		QUANTITY: 22 PCS EST.		TRUCK B/L #: 2068433		GAUGE: 0.2500 IN (6.3500 MM)		DIAMETER/WIDTH: 48.000 IN (1473.2 MM)		LENGTH: 144.500 IN (3670.3 MM)			

MHU 2142043: LOT 231987B9: 22 pieces;

Certified Specifications

AMS 4037/RevQ

AMS-QQ-A-250/4/RevB

ASTM B 209/Rev14

Test Code: 1504

Lot: 231987B9

Cast 493

Drop 59

Test Results

Ingot 2

Melted in USA

(ASTM E8/B557)

(EN 2002-1)

Tensile:	Temper	Dir / # Tests	Ultimate KSI (MPA)	Yield KSI (MPA)	Elongation %
	T351	LT / 2 (Min:Max)	70.4 : 70.6 (485 : 487)	48.6 : 48.6 (335 : 335)	18.6 : 19.9

(ASTM E1251)

Chemistry:	SI	FE	CU	MN	MG	CR	ZN	TI	V	ZR	OTHER
Actual(wt%)	0.06	0.17	4.7	0.59	1.4	0.01	0.09	0.02	0.01	0.01	TOT 0.06

ALLOY LIMITS

	SI	FE	CU	MN	MG	CR	ZN	TI	V	ZR	OTHER	MAX
2024												
MIN(wt%)	0.00	0.00	3.8	0.30	1.2	0.00	0.00	0.00	0.00	0.00	EACH	0.05
MAX(wt%)	0.50	0.50	4.9	0.9	1.8	0.10	0.25	0.15	0.05	0.05	TOT	0.15

Aluminum Remainder