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# COMDES-MELT: A Turbofan Engine Icing Risk Analysis Tool, User's Manual

*Philip C. E. Jorgenson and Joseph P. Veres  
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An Erratum was added to this report September 2020.

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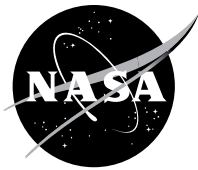
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National Aeronautics and  
Space Administration

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# Erratum

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 Glenn Research Center, Cleveland, Ohio

March 2020

This Technical Memorandum is the user’s manual for the releasable version of the code, which is available on the NASA Software Repository, <https://software.nasa.gov/>.

Page 83, Section 7.4.1.1 Input File: Replace data with the following.

**comdes.dat**

```

Single Centrifugal SRW HPC Option # 2 Low Ns 1-20-2010
30.0 159.25 1119.39 38620.0 Wact Pt1 Tt1 RPM
5 1 0 0.00 Fluid Stages IDUMMY DUMMY
0.00 46.900 000.00 0.00 DropDia InDist RHinlet IWC
***icing***
2.70 0.00 406.44 0.00 Psin DUMMY Tsin DUMMY
***icing***
607.8 0.00 00000.00 0 Vin DUMMY Altitude IDUMMY
***icing***
3.40 2.41 5.88 5.88 1 Rltip Rlhub R2tip R2hub
0.3300 0.380 30 0.0300 1.0 Blaxial B2axial NBLAD THK
CFS 0.0 0.0000 0.000 1.0000 1.0 StatorT StatorM StatorH FSGV
CFMid 0.9800 0.9500 1.1000 0.9000 1.0 AeroB1 AeroB2 POTS POTH
CFHub
0.842 0.9220 0.917 0.0000 EfficiT EfficiM EfficiH BLEED
0.00 0.00 0.00 0.00 DUMMY DUMMY DUMMY DUMMY
0.8500 0.16 1.460 0.00 SlipF DiffLoss Solidity RPM2
51.50 50.60 48.00 0.0 BetaBlade1T BetaBlade1M BetaBlade1H expo3
23.50 23.50 23.50 2 BetaBlade2T BetaBlade2M BetaBlade2H stat2
6.35 6.35 0.38 67.5 R3s R3h B3X BetB3
74 0.03 0.95 0.01 Blan3 Blath3 Blok3 Wbar23
6.60 10.60 0.95 0.015 RCG Athrt BlokTH Wbar2T
9.55 9.55 0.37 55.0 R4S R4H B4X BetB4
74 0.03 0.95 1.30 Blan4 Blath4 Blok4 SolidStat
10.50 10.10 0.00 50.0 R5s R5h B5X BetB5
95 0.03 0.95 0.01 Blan5 Blath5 Blok5 Wbar56
0.00 0.000 0.95 0.015 RCG2 Athrt2 BlokTH Wbar5T
10.50 10.10 0.0 0.0 R6S R6H B6X BetB6
95 0.03 0.95 1.30 Blan6 Blath6 Blok6 SolidStat
10.50 10.10 0.00 1.0 R7S R7h B7X Length67
2 0.01 0.98 0.000 Blan7 Blath7 Blok7 Wbar67
10.50 10.10 0.0 1.0 R8S R8h B8X Length78
2 0.01 0.98 0.00 Blan8 Blath8 Blok8 Wbar78

```

Page 85, Section 7.4.1.2 Output File: Insert “Overall Exit Conditions” data.

```

OVERALL EXIT CONDITIONS; ALL          1  STAGES

Del Enthal  DelHT/U1^2  GHP      MassFloSlcor  OPR      Efficiency  RotorlInc  TR      AxHubLen
2935275.00  2.2355  4976.2515  4.068        2.5715  0.71662    6.0454    1.3959  5.673

```



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# **COMDES-MELT: A Turbofan Engine Icing Risk Analysis Tool, User's Manual**

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## **Summary**

The computational tool COMDES-MELT was developed to predict the susceptibility of turbofan engines to ice accretion when flying in an ice crystal environment. COMDES-MELT is a first-generation computational tool that can estimate the conditions at which the accretion of ice can occur, location within the turbofan engine compression system, and at which engine operating points within the flight trajectory. This is accomplished by tracking the parameters of static wet bulb temperature, ice particle melt ratio, and the ice-water to airflow rates ratio as ice crystals are ingested into the engine compression system. It has been used successfully to predict engine operating points for simulation in an altitude wind tunnel with ice crystal ingestion. These engine tests occur in a simulated ice crystal cloud environment. This predictive analysis has been used to generate the test matrix in preparation for tests in the Propulsion Systems Laboratory, an altitude test facility at the NASA Glenn Research Center. Furthermore, due to the speed of the code during an engine icing test, adjustments to the test matrix can be provided to the test engineer in real time to focus on a particular area of ice accretion susceptibility. The use of the COMDES-MELT code will be examined in this manual.

## **1.0 Introduction**

Commercial airlines flying through clouds with high ice water content have experienced engine icing events (Ref. 1). These events have been attributed to ice crystal ingestion into the turbofan engine, which results in accretion on the components of the compression system. The total number of engine icing events per year appears to be increasing since 2002 (Ref. 2). The engine events have occurred at altitudes between 10,000 and 40,000 ft and at temperatures between the International Standard Atmosphere (ISA) and 36 °R above ISA temperature. As ice crystals are ingested into the fan and low-pressure compression system, the air static temperature typically increases and a portion of the ice crystals melt due to the warmer air. This allows the ice-water mixture to stick and accrete upon impacting the metal surfaces of the compressor components. It is hypothesized that this allows the ice-water mixture to cover the metal surfaces of the compressor stationary components, which leads to ice accretion through evaporative cooling. The accreted ice can cause the occurrence of one or more of the following failure modes: uncommanded loss of thrust control (engine rollback), compressor surge or stall, ice shedding resulting in structural damage to the downstream compressor blades, and possible combustor flameout.

A computational tool has been developed and applied to the compression systems of turbofan engines to predict the locations where there is an ice accretion risk at various altitudes and operating conditions in a flight trajectory. The primary purpose of this user's manual is to demonstrate the use of this icing risk prediction tool.

## 2.0 Key Features

The primary focus of this user's manual is to describe the COMDES (Ref. 3) mean-line compressor flow analysis computer code and its effective implementation and use as an engine icing risk prediction tool COMDES-MELT (Refs. 4 and 5). This analysis tool has the capability of computing the risk of ice accretion in the turbofan engine compression system by tracking key parameters through the compression system blade rows at all engine operating points within the flight trajectory. The results from an engine system thermodynamic cycle code or experimental data is required to provide boundary conditions to the COMDES code, which together with an ice particle melt code, MELT, has the capability of determining the rate of sublimation, melt, and evaporation of ice particles as they travel through the compressor blade rows.

The COMDES-MELT code computes the velocity, pressure, temperature, and flow angles at the leading and trailing edges of each blade row and at the hub, mean, and tip sections. This compressor code includes the ability to calculate the effects of water vapor on the fluid properties of the air-water vapor mixture based on the mole fraction of air to water vapor. The relative humidity at the engine inlet is specified, and the local specific humidity (mass of water to mass of air) is computed through each component of the inlet-fan low-pressure compressor (LPC), taking the sublimation, melt, and evaporation into consideration, as well as the local air temperature. The resulting effect of the humid air on the performance of the compressor is computed. Several key parameters have been identified as early indicators of an ice accretion risk: the local wet-bulb temperature, the melt ratio within each blade row, and the ice water to airflow rate ratio (IWAR). The local wet-bulb temperature is calculated at each blade row. The geometry section of the code is defined by simple circular arc rotors and stators, including stage axial gaps. As the ice particles pass through the compression rotors, stators, and gaps, the ice particle melting and evaporation model computes the local melt ratio, change in enthalpy, and particle temperature and diameter through each compressor component. The ratio of IWAR is computed at the inlet based on the ice water content in the atmosphere and the mass flow rate of air entering the engine. If the limiting values of these key parameters are met, there is a risk that ice will accrete on the surfaces of the compressor. With these parameters as the precursors to ice accretion, the blade row within the compression system can be identified that is likely to experience ice buildup at a particular engine operating condition in the vehicle flight trajectory.

Assumptions are made in predicting the complex physics involved in engine icing. Specifically, the code does not directly estimate ice accretion and does not have models for ice particle breakup or erosion. The code is used to determine the risk of icing by computing engine operating points in the flight trajectory over a range of atmospheric ice conditions (discrete ice particle sizes and concentrations). The computational tool can be used to assess specific turbofan engines as to their susceptibility to ice accretion in an ice crystal environment. Without additional information about the wall heat transfer, this code is limited to adiabatic flow paths.

In the following sections, this user's manual includes a general description of the analysis code and its current capabilities. The code execution, input and output file variable descriptions, and example cases, with their output and corresponding input files, are described as well. The example section includes information on how an input file is created and the sensitivity of the rotor exit flow angle, pressure, and temperature to the slip factor, blockage, and efficiency specified as parameters. In addition, a stage-matching example is provided, which illustrates how a single stage can be the limiting component in the overall performance of the compressor. The COMDES code is capable of analyzing axial, centrifugal, and mixed axial and centrifugal compressors. Though the icing analysis has been computed only on axial compressors, examples of centrifugal and mixed axial and centrifugal compressor analysis without ice particle ingestion are included for completeness.

### 3.0 COMDES-MELT: Engine Icing Risk Prediction Tool

The engine icing risk prediction code COMDES-MELT is a mean-line compressor analysis code coupled with an ice crystal thermodynamic state code. A general description of the mean-line code will be given followed by specifics of how the enthalpy exchange between ice particles and the air is incorporated into the flow analysis. In a following section, an example is outlined where a conceptual design is generated. Its performance is predicted and compressor maps are generated. It is then further analyzed for icing risk.

The COMDES code is based on the compressible fluid flow equations and the Euler equation. The rotor efficiency and stator (or diffusion system) losses at the design point are input items into the code. The design point rotor efficiency is modeled at offdesign by an empirical correlation with rotor incidence. The relative velocity ratio and the diffusion factor through the rotor are correlated to the onset of stall. The equations and variables used in the mean-line analysis are listed in the governing equations section of this user's manual.

#### 3.1 Flow Path and Velocity Triangle Definitions

COMDES is a compressor analysis code and requires specific input parameters: geometric, aerothermodynamic, and kinematic. The rotor hub and tip radii from the centerline that define the flow path are input at the leading and trailing edges as shown in Figure 1. Other input parameters are the inlet total pressure and temperature, mass flow, shaft rotational speed, rotor aerodynamic blockages at the inlet and exit, efficiency, and slip factor.

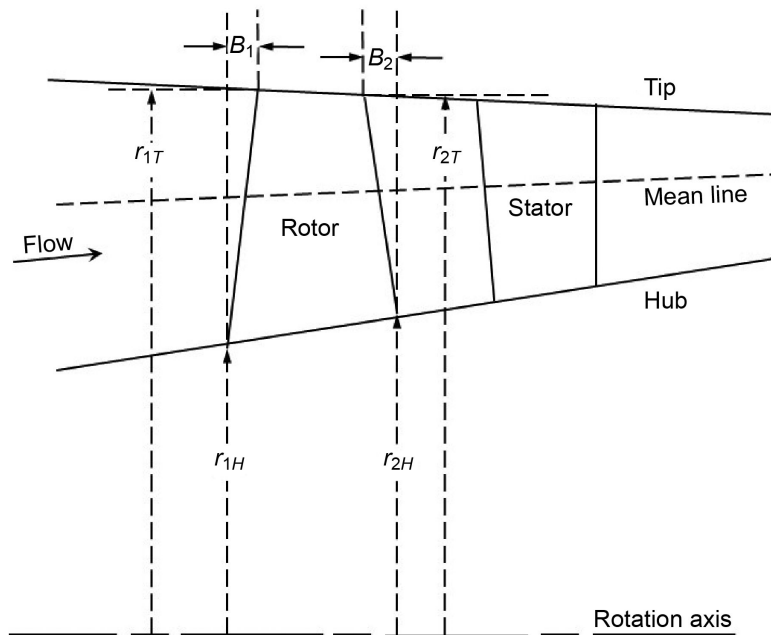


Figure 1.—Axial compressor flow path geometry input parameters to COMDES. Edge sweep is  $B$ ; blade radius is  $r$ . Subscripts: leading edge is 1; trailing edge is 2; hub is  $H$ ; tip is  $T$ .

The code computes the pressure and temperature rise through the rotor. The mass flow is then varied through the compressor along a constant speed line to determine flow conditions that result in choke or surge. Choke is determined as the limit of flow where small increases in corrected mass flow results in large drops in calculated pressure ratio, and eventually, the code will fail to converge on the rotor exit velocity triangles. Surge has been determined empirically from the NASA Stage 37 and the 74-A compressor data correlated to the values of rotor diffusion factor and relative velocity ratios at the tested surge point (Ref. 3).

At each point along each speed line, the iteration begins with solving for the static pressure and temperature at the rotor leading edge, based on the available area, incidence, mass flow rate, and the inlet total pressure and temperature. Once the inlet Mach number and static pressure and temperature have been computed, all of the components of the absolute and relative velocities and flow angles are determined as shown in Figure 2.

The rotor exit deviation angle is taken into consideration in the iterative calculations and is specified as an input item in terms of a slip factor as defined by Equation (28) in the next section. The code calculates all of the components of the rotor exit velocity diagram shown in Figure 2 by solving Equations (20) to (36), presented in Section 4.3, simultaneously using an iterative technique. All of the components of velocity at the rotor trailing edge are derived from the enthalpy rise as calculated from Equations (21) and (22) in terms of temperature rise across the rotor, and the Euler equation. An analogous illustration of the inlet and exit velocity diagrams for the centrifugal impeller is included in Example Case (4) (see Figure 26).

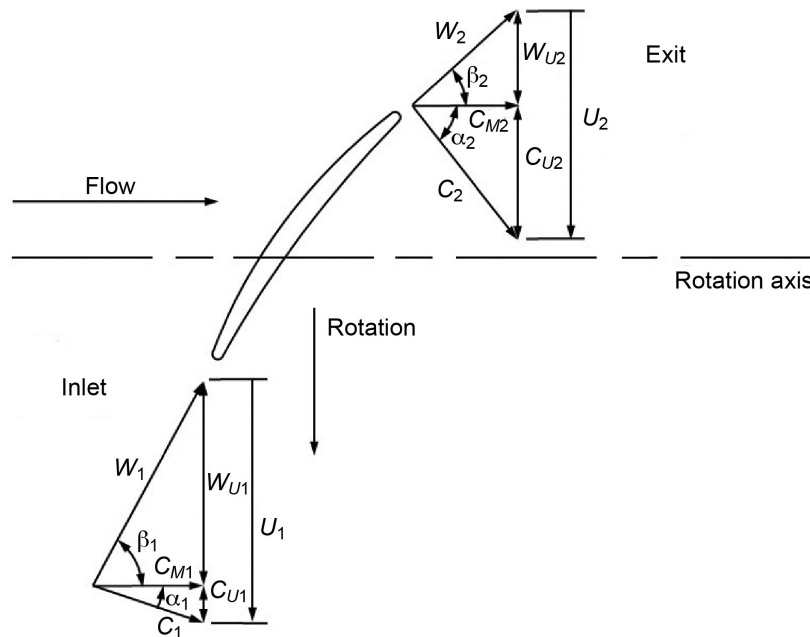


Figure 2.—Axial rotor inlet and exit velocity triangles. Absolute velocity is  $C$ ; rotor peripheral velocity is  $U$ ; relative velocity is  $W$ ; absolute flow angle is  $\alpha$ ; relative flow angle is  $\beta$ . Subscripts: leading edge is 1; trailing edge is 2; meridional is  $M$ ; tangential is  $U$ .

The mean-line computer code is executed to estimate the performance of the compressor at off-design, or part-speed, operation. To determine the efficiency at off-design conditions, the compressor rotor efficiency at the design point is input. Note that the design point efficiency is that value only at a rotor incidence of  $6^\circ$  just ahead of the rotor, not including metal blockage at the leading edge. This design point incidence was also determined empirically from analysis of test data (Ref. 3). The design point efficiency is assumed to be the same along all speed lines, and is varied only as a function of rotor incidence angle. The losses through the stators due to incidence are not varied directly but are lumped into the efficiency calculation, which is a function of rotor incidence as defined by Equation (36). This is a simplification which assumes that as the flow rate changes along a speed line, both the rotor and the stator experience a change of incidence from their design value, and therefore, a reduction in efficiency occurs. A model for stator loss variation with incidence that is separate from the efficiency deration due to incidence of the rotor can be included in a future version of the code. Also, the stator and rotor total-pressure loss coefficients need to be replaced in a future version of the code by a better correlation as a function of Mach number and incidence.

### 3.2 Rotor and Stator Blade Geometry Generator

The geometry generator produces compressor blades that are in the shape of a circular arc. The code generates rotor blades and stator vanes based on the values of the blade angles at the leading and trailing edges, the tip solidity, and the flow path radii. The circular arc-shaped mean camber lines of the blades are created at the hub, mean, and tip section of each rotor and stator of the stage. The compressor blade shapes used in the analysis are based on the circular arcs, which act as reasonable approximations of the actual rotors and stators (Figure 3). This represents one stage of the multistage fan core and LPC. With this method, the blade-to-blade distances and passage chord lengths can be computed with reasonable accuracy. The lengths are used to estimate ice particle residence time which is required for calculating the sublimation, melt, and evaporation rates.

The tip solidity, number of blades, and flow path inner and outer radii are input parameters. The simple circular arc blade camber line is generated by the code at the hub, mean, and tip sections. The same procedure is followed to provide the stator vane camber line with simple circular arc shape. The following equations define the rotor blade and stator vane (referred to as airfoils in the following) camber geometric shape. The chord,  $\bar{C}$ , is determined from the tip solidity,  $\sigma$ , the tip radius,  $r_T$ , and the number of airfoils,  $N$ , by Equation (1).

$$\bar{C} = \frac{2\pi r_T \sigma}{N} \quad (1)$$

The true chord can be closely approximated by the following equation, which is for the isosceles triangle illustrated in Figure 4 where  $\theta$  is the blade arc angle.

$$\bar{C} = 2r_c \sin \frac{\theta}{2} \quad (2)$$

The following equations are used to solve for the coordinates of the airfoil leading and trailing edges and the coordinates of the center of the circular arc, which defines the arc radius  $r_c$  at the blade midchord. The leading and trailing edge blade metal angles are defined by  $\beta_{B1}$  and  $\beta_{B2}$ , respectively.

$$r_c^2 = X_1^2 + Y_1^2 \quad (3)$$

$$r_c^2 = X_2^2 + Y_2^2 \quad (4)$$

$$\sin \beta_{B1} = \frac{X_1}{r_c} \quad (5)$$

$$\cos \beta_{B2} = \frac{Y_2}{r_c} \quad (6)$$

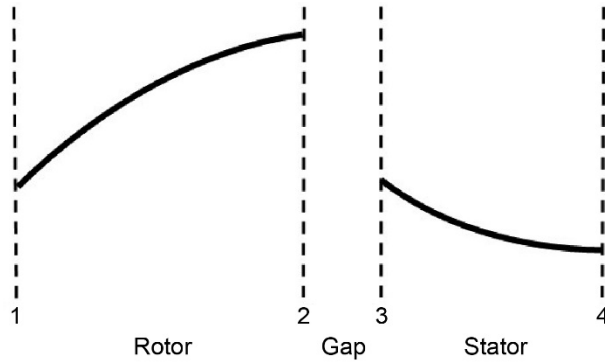


Figure 3.—Rotor and stator blade mean camber line at the midspan section for a typical stage. The station numbers 1 and 2 refer to the rotor leading and trailing edges, respectively, while stations 3 and 4 refer to the stator leading and trailing edges, respectively.

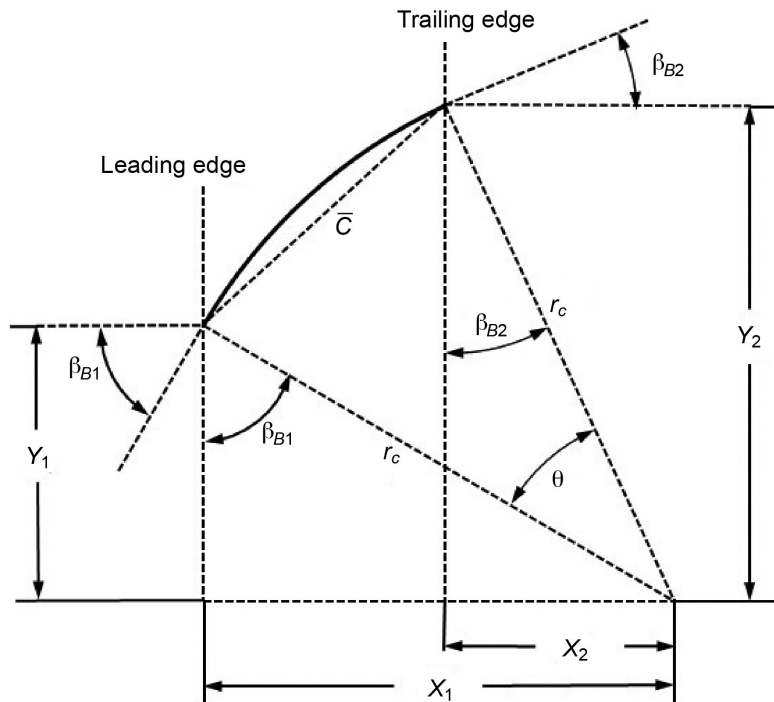


Figure 4.—Compressor airfoil camber line geometry creation in COMDES. Simple circular arc camber line with radius  $r_c$  is generated in compressor code based on input values of tip solidity of rotor and stator, and metal angles at leading and trailing edges  $\beta_B$ . Where chord is  $\bar{C}$ ,  $\theta$  is blade arc angle, and subscripts 1 and 2 are leading and trailing edges, respectively.

The airfoil section coordinates shown in Figure 4 are determined using the previous equations for a circular arc. Both the rotor blades as well as the stator vanes are generated with a single circular arc. The geometric data previously described can be found in the geometry file (geometry.out) generated when the COMDES-MELT code is executed. The rotor blade and stator vane circular arc camber line geometries at the hub, mean, and tip are written to the file Plot\_Geometry.dat and can be adapted for any plotting application.

### 3.3 Governing Equations

The rotor annular area  $A$  is calculated from the hub and tip radii from the centerline by Equation (7) with  $r$  and  $B$  defined as illustrated in Figure 1. The aerodynamic blockages,  $\lambda$ , are specified as input parameters at the leading and trailing edges of the rotor in the mean-line analysis, and are not varied at off-design operating conditions.

$$A = \lambda \pi (r_T + r_H) \left[ (r_T - r_H)^2 + B^2 \right]^{1/2} \quad (7)$$

The static temperature  $T_S$  is a function of the total temperature  $T_t$  and the Mach number  $M$  with  $\gamma$  being the ratio of specific heats as shown in Equation (8).

$$\frac{T_t}{T_S} = 1 + \frac{\gamma - 1}{2} M^2 \quad (8)$$

The static pressure  $P_S$  is a function of the total pressure  $P_t$  and  $M$  as shown in Equation (9).

$$\frac{P_t}{P_S} = \left( \frac{T_t}{T_S} \right)^{\frac{\gamma}{\gamma - 1}} = \left( 1 + \frac{\gamma - 1}{2} M^2 \right)^{\frac{\gamma}{\gamma - 1}} \quad (9)$$

The volume flow rate  $Q$  into the rotor is determined by Equation (10) in terms of the mass flow rate  $m$ ,  $P_S$ , and  $T_S$ .

$$Q = \frac{m T_S R}{P_S} \quad (10)$$

The absolute meridional velocity  $C_M$  at the rotor inlet is shown in Equation (11) as a function of  $Q$  and inlet area.

$$C_M = \frac{Q}{A} \quad (11)$$

The tangential component of absolute velocity  $C_U$  is determined from the  $C_M$  and the inlet swirl angle  $\alpha$  as shown in Equation (12).

$$C_U = C_M \tan(\alpha) \quad (12)$$

The absolute total velocity  $C$  is determined from the  $C_M$  and the  $C_U$  as shown in Equation (13).

$$C = (C_M^2 + C_U^2)^{1/2} \quad (13)$$

The speed of sound  $a$  is determined from the  $T_S$  and the gas properties, universal gas constant  $\mathfrak{R}$  and conversion factor  $g_c$ , as shown in Equation (14).

$$a = (\gamma \mathfrak{R} T_S g_c)^{1/2} \quad (14)$$

The  $M$  is determined from  $C$  and  $a$  as shown in Equation (15).

$$M = C/a \quad (15)$$

The flow coefficient of the rotor  $\Phi$  is defined by Equation (16) in terms of the axial component of velocity and the rotor peripheral tip speed  $U_T$ .

$$\Phi = \frac{C_M}{U_T} \quad (16)$$

The relative flow angle  $\beta_F$  is determined by Equation (17) where  $U$  is the rotor peripheral velocity.

$$\beta_F = \tan^{-1} \left[ \frac{U - C_U}{C_M} \right] \quad (17)$$

The blade metal angle  $\beta_B$ , flow angle  $\beta_F$ , and incidence  $i$  for a given relative velocity,  $W$ , are illustrated in Figure 5. The rotor incidence angle  $i_R$  is defined as the difference between the relative flow angle and the blade angle at the rotor leading edge just upstream of the blade by Equation (18).

$$i_R = \beta_F - \beta_B \quad (18)$$

The relative velocity at the rotor leading edge is determined by Equation (19).

$$W = \frac{C_M}{\cos(\beta_F)} \quad (19)$$

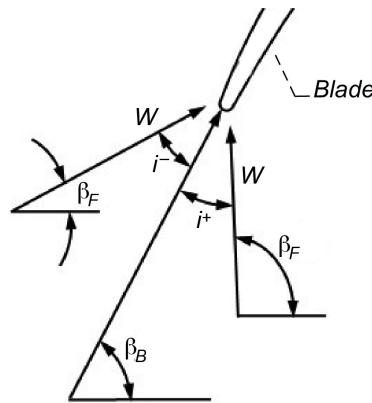


Figure 5.—Axial compressor rotor leading edge blade metal angle  $\beta_B$ , flow angle  $\beta_F$ , and incidence  $i$ . Relative velocity is  $W$ .



The rotor total-to-total adiabatic efficiency  $\eta$  is defined by Equation (20).

$$\eta = \frac{\left(\frac{P_{t2}}{P_{t1}}\right)^{\frac{\gamma-1}{\gamma}} - 1}{\frac{T_{t2}}{T_{t1}} - 1} \quad (20)$$

The enthalpy rise  $\Delta H$  is calculated in terms of temperature rise across the rotor by Equation (21) where  $J$  is a units conversion and  $c_p$  is the specific heat of the gas and the Euler Equation (22), which is based on the change in tangential velocity from inlet to exit.

$$\Delta H = J c_p g_c (T_{t2} - T_{t1}) \quad (21)$$

$$\Delta H = U_2 C_{U2} - U_1 C_{U1} \quad (22)$$

The  $C_M$  at the rotor trailing exit is determined from the density, mass flow rate, and area as shown in Equation (23).

$$C_{M2} = \frac{m R T_{S2}}{P_{S2} A_2} \quad (23)$$

The relative velocity at the rotor exit is determined from Equation (24).

$$W_2 = \left[ (U_2 - C_{U2})^2 + C_{M2}^2 \right]^{1/2} \quad (24)$$

The absolute flow angle at the rotor exit is determined from the relation shown in Equation (25).

$$\alpha_2 = \tan^{-1} \left( \frac{C_{U2}}{C_{M2}} \right) \quad (25)$$

The  $C$  at the rotor exit is determined from Equation (26).

$$C_2 = (C_{U2}^2 + C_{M2}^2)^{1/2} \quad (26)$$

The tangential component of relative velocity at the rotor exit is determined from Equation (27).

$$W_{U2} = (W_2^2 - C_{M2}^2)^{1/2} \quad (27)$$

The slip factor  $\delta$  is the difference between the theoretical and absolute fluid tangential velocities normalized with blade peripheral speed as described by Equation (28). The slip factor is an input item and is typically from 0.93 to 0.96 for axial compressors. For centrifugal compressors, the slip factor can be on the order of 0.80 to 0.90, depending on number of blades and exit blade angle.

$$\delta = \frac{C_{U2} + C_{M2} \tan(\beta_{B2})}{U_2} \quad (28)$$

The relative flow angle at the rotor exit is determined from Equation (29).

$$\beta_{F2} = \tan^{-1} \left[ \frac{(U_2 - C_{U2})}{C_{M2}} \right] \quad (29)$$

The rotor diffusion factor  $DF$  at the root-mean-square radius is calculated by Equation (30).

$$DF = 1 - \frac{W_2}{W_1} + \frac{(r_1 C_{U1} - r_2 C_{U2})}{(r_1 + r_2) W_1 \sigma} \quad (30)$$

The incidence at the stator vane leading edge  $i_V$  can be estimated by Equation (31).

$$i_V = \alpha_2 - \beta_V \quad (31)$$

The stator loss coefficient  $\omega_V$  is defined by Equation (32).

$$\omega_V = \frac{P_{I4} - P_{I2}}{P_{I2} - P_{S2}} \quad (32)$$

The power,  $\bar{P}$ , required to drive the compressor is calculated from Equation (33).

$$\bar{P} = \frac{\Delta H m}{550 g_c} \quad (33)$$

The inlet guide vane setting angle versus loss of total pressure that is used by the code was determined empirically from the NASA 74–A compressor, which has a flap-style variable inlet guide vane, and is shown in Equation (34). As such, these losses are considered a good approximation for the 76–B compressor, which also has similar flap-style inlet guide vanes. Note that these losses would likely be different for other types of variable guide vane configurations and may not provide an accurate representation of losses.

$$\Delta P_t = 0.0059374 - 0.00055463747S + 0.00007424S^2 + 0.000000458487S^3 \quad (34)$$

The total pressure at the exit of the variable inlet guide vane is obtained from Equation (35).

$$P_{t\text{exit}} = P_{t\text{inlet}} (1 - \Delta P_t) \quad (35)$$

The efficiency of the rotor as a function of incidence, shown in Equation (36), was determined empirically from the Stage 37 and 74–A compressors analyzed in this report and is currently the model in the compressor code for estimating the off-design efficiency. This same model is used on all speed lines for rotor efficiency deration versus incidence.

$$\Delta \eta = 0.0006i_R^3 - 0.0185i_R^2 + 0.1699i_R + 0.5187 \quad (36)$$

## 4.0 Engine Icing Risk Prediction

In order to perform an engine icing study, it is necessary to have the geometry of the compressor (e.g., LPC) in adequate detail such that the flow conditions between each blade row can be computed with the mean-line compressor flow code. This data can be obtained from a streamline curvature code, conceptual design, or from experimental data. For this report, the data is extracted from a conceptual design.

#### 4.1 An Example for Fan and Low-Pressure Compressor With Conceptual Design

The conceptual design task was performed for a notional turbofan engine in the 40,000 lbf thrust class having a bypass ratio of 5.8 with the mean-line compressor flow analysis code. The rotor and stator blade shapes were approximated with simple circular arcs. The conceptual design effort of the notional fan core and LPC determined the key dimensions that define the blade and flow path geometry. Figure 6 illustrates the notional fan and LPC that is used as an example in this user’s manual.

The overall design point objectives for the full fan, fan tip, and fan core with LPC are described in Table I. The conceptual sizing study of the fan and LPC flow path was completed with the COMDES mean-line compressor flow analysis code. The resulting flow path showing the rotor blade and stator vane leading and trailing edges is illustrated in Figure 6. The key geometric and design point aerodynamic details of the fan core and LPC resulting from the mean-line conceptual design are summarized in Table II. This conceptual design will be used to conduct an icing risk analysis of the resulting notional fan and LPC at an operating point in a flight trajectory.

TABLE I.—FAN AND LOW-PRESSURE COMPRESSOR (LPC) DESIGN POINT OBJECTIVES

Fan, full	
Flow, corrected, lbm/s.....	1,033.6
Pressure ratio.....	1.40
Efficiency, percent.....	89.6
Shaft speed, corrected, RPM.....	3,761.1
Fan, tip	
Flow, corrected, lbm/s.....	861.2
Pressure ratio.....	1.426
Efficiency, percent.....	90.0
Shaft speed, corrected, RPM.....	3,761.1
Fan-core and LPC	
Flow, corrected, lbm/s.....	172.3
Pressure ratio.....	2.23
Efficiency, percent.....	87.0

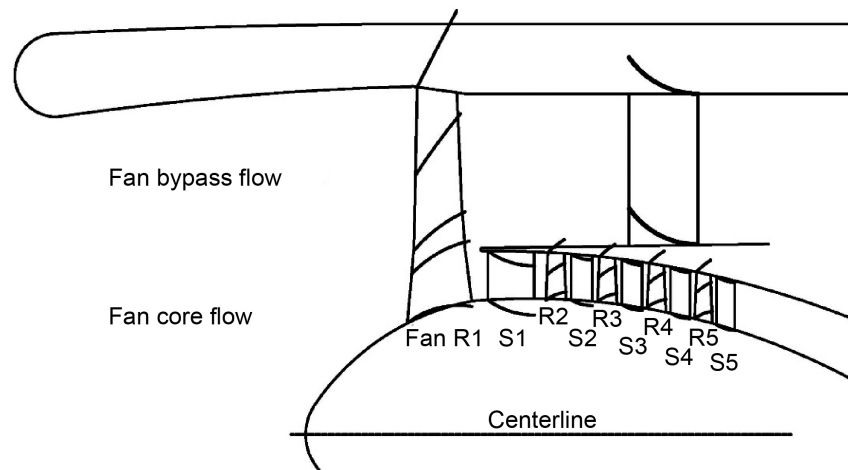


Figure 6.—Fan and low-pressure compressor of notional high bypass ratio turbofan engine.

TABLE II.—FAN CORE AND LOW-PRESSURE COMPRESSOR (LPC)  
DESIGN POINT GEOMETRIC AND AERODYNAMIC PARAMETERS

Parameter	Fan-hub rotor 1	Fan-hub stator 1	LPC rotor 2	LPC stator 2	LPC rotor 3	LPC stator 3	LPC rotor 4	LPC stator 4	LPC rotor 5	LPC stator 5
Leading edge										
Pressure static, psia	13.67	14.67	16.61	18.785	20.12	22.45	23.91	26.41	27.88	30.04
Temperature static, R	508.0	522.1	542.22	564.55	576.56	597.76	609.40	629.48	640.0	656.21
Mach no., abs	0.38	0.638	0.46	0.534	0.42	0.499	0.39	0.449	0.34	0.417
Blade angle, deg	47.20	35.4	42.3	30.6	23.5	31.5	43.80	32.4	44.80	33.0
Absolute flow angle, deg	0	36.15	0	31.22	0	32.18	0	33.00	0	0
Tip radius, in.	20.63	20.6171	20.5304	20.3433	20.0960	19.6905	19.3131	18.8420	18.2524	17.6890
Hub radius, in.	12.51	15.1021	15.2113	15.2026	15.0546	14.8484	14.3197	13.7204	13.0749	12.2606
Trailing edge										
Pressure static, psia	14.69	16.35	18.76	19.79	22.40	23.56	26.19	27.53	30.06	31.22
Temperature static, R	521.86	539.72	563.96	573.81	597.00	606.82	627.60	637.69	655.84	663.90
Mach no., abs	0.63	0.481	0.53	0.448	0.49	0.413	0.46	0.368	0.41	0.338
Blade angle, deg	12.7	0	23.5	0	23.5	0	23.5	0	23.5	0
Absolute flow angle, deg	36.93	0	31.59	0	32.30	0	32.27	0	33.24	0
Tip radius, in.	20.63	20.5733	20.4459	20.1704	19.9256	19.4117	19.0737	18.4850	17.9280	17.2148
Hub radius, in.	15.0	15.1901	15.2230	15.2026	14.8484	14.4742	13.9111	13.2848	12.4811	11.6228

## 4.2 Performance of Notional Turbine Engine System Model Through Typical Commercial Aircraft Flight Trajectory

In order to perform the engine icing study, the COMDES engine model was executed at numerous operating conditions through a flight trajectory. The inlet conditions of ice crystal concentrations in  $\text{g}/\text{m}^3$ , particle size, and the elevated ISA temperature were varied parametrically between 0 and 36 R, since that is in the range of where engine icing events have been reported (Refs. 1 and 2). The detailed analysis of the fan core and LPC with the COMDES code was obtained at each operating condition. A system model (Ref. 6) was utilized to determine the performance of the notional 40,000 lbf thrust class engine through a typical flight trajectory of a commercial aircraft (Figure 7).

The notional flight trajectory (Refs. 4 and 7) used in this study includes takeoff, climb, two cruise altitudes, and one descent profile. The first cruise altitude is at 35,000 ft, where the engine thrust is reduced from 100 percent down to 69.9 percent of maximum thrust at the end of the cruise. The aircraft then proceeds to 39,000 ft where again the thrust reduces from 100 percent down to 76.8 percent maximum thrust at that altitude. The engine is then spooled back to 10 percent maximum thrust at 39,000 ft as it prepares to descend in altitude. The engine is maintained at 10 percent maximum thrust as it descends to 10,000 ft, which was the lowest altitude that was analyzed in this study. The ambient temperature at 10,000 ft altitude was considered to be above that which would support ice crystals in the atmosphere.

Figure 8 and Figure 9 illustrate the baseline pressure ratio and efficiency characteristic maps of the fan core and LPC and the superimposed operating points through the flight trajectory. The baseline maps have no additional blockage due to ice accretion. These maps were generated by the compressor code prior to any ice crystal ingestion. The engine icing risk compressor flow analysis code, COMDES-MELT, is then executed with ice crystal ingestion (particle size and concentration) at specific operating points within the flight trajectory to determine if there is a risk of ice accretion.

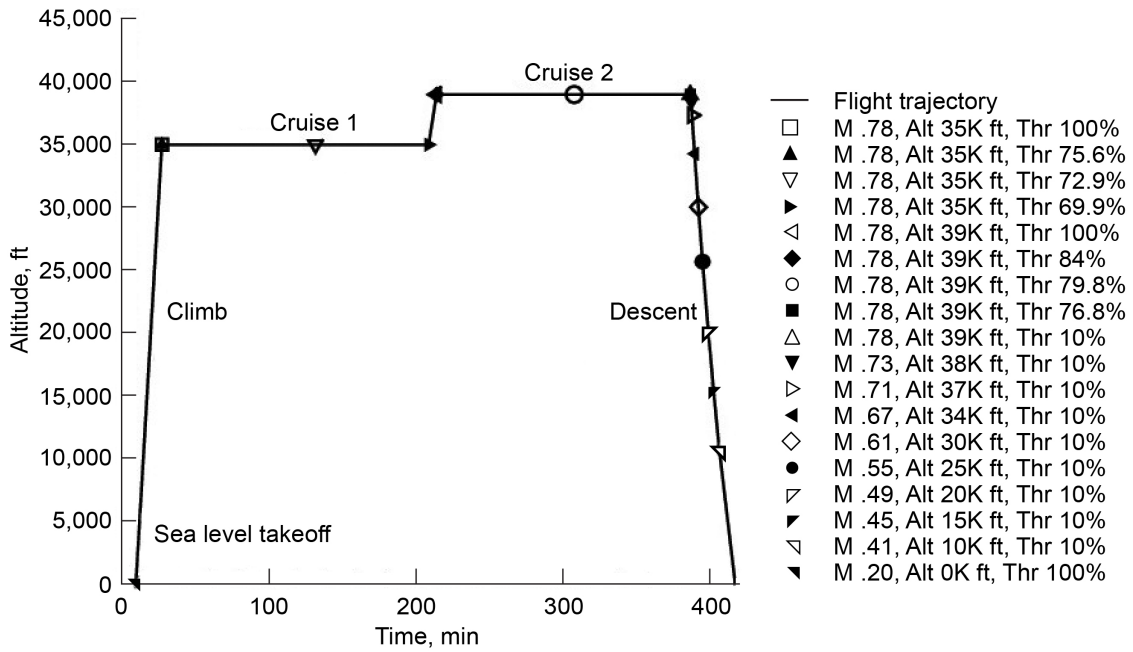


Figure 7.—Flight trajectory for typical commercial aircraft. The notional 40,000 lbf thrust engine is “flown” through vehicle flight trajectory to determine blade row where conditions for ice accretion may be possible.

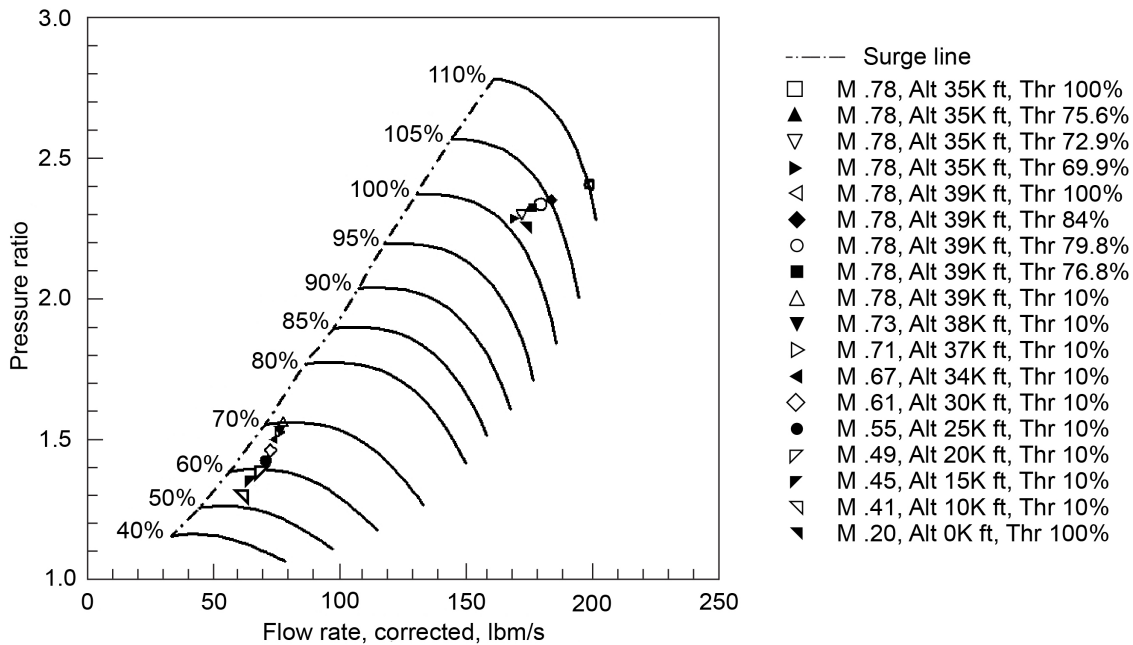


Figure 8.—Fan core and low-pressure compressor (LPC) pressure ratio showing baseline performance. Operating points of fan core and LPC throughout flight trajectory are superimposed onto baseline pressure ratio map.

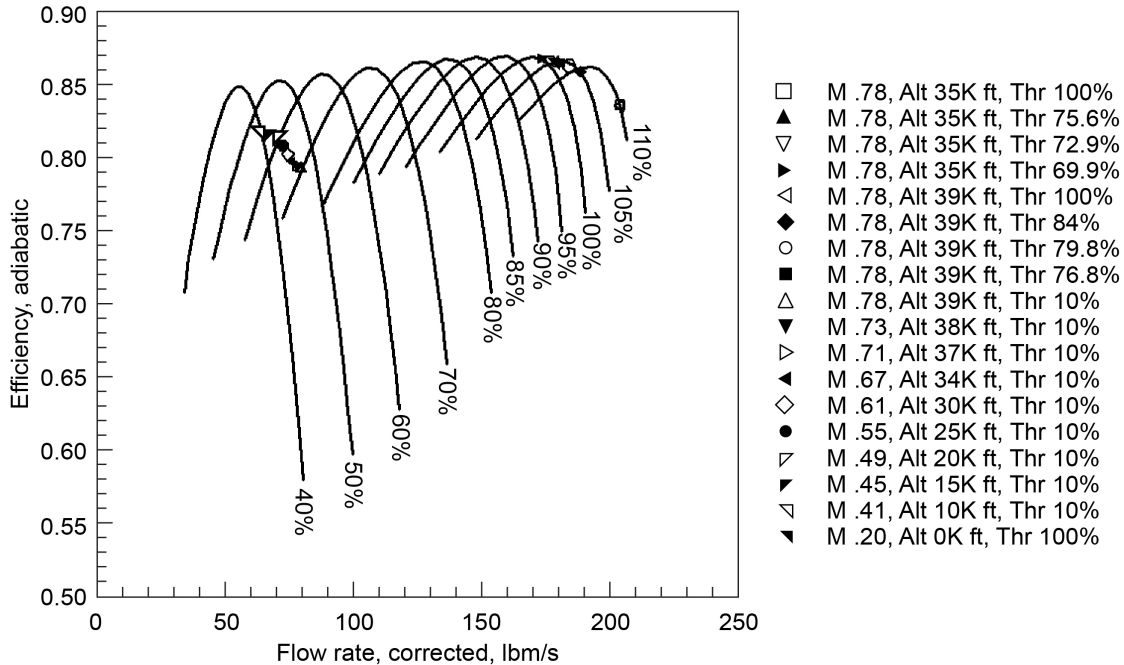


Figure 9.—Fan core and low-pressure compressor (LPC) performance showing baseline efficiency map. Operating points of fan core and LPC throughout flight trajectory are superimposed onto baseline efficiency maps.

The risk of ice accretion in the compression system is a function of location in the flight trajectory where the engine is operating and the atmospheric conditions of temperature and pressure, as well as the ice particle concentration and size.

A range of ice particle concentrations and diameters were analyzed with the icing risk code, at various ambient temperatures above the ISA standard. The performance of the fan and LPC stages were analyzed in terms of the local static wet-bulb temperature ( $T_{WBS}$ ) and the particle melt ratio. As the engine is operated through the flight trajectory, the local values of wet-bulb temperature ( $T_{WB}$ ) at each blade and stator row are calculated with the compressor flow code. It is assumed that ice accretion can only occur at values of  $T_{WBS}$  that are below the temperature of 498 R. This assumption is based on observations (Refs. 8 and 9) that showed significant growth rates and buildup of ice at values of  $T_{WBS}$  between 492 and 498 R. It was also found that the IWAR was needed to support ice accretion. Values of IWAR in general need to be above 0.002 for ice accretion to occur. As an example, the distribution of  $T_{WBS}$  and particle melt ratio are provided (see Figure 13), illustrating how each of these parameters are computed for one operating point that indicates a risk for this notional engine.

### 4.3 Energy Balance Between Ice Particle and Air

As the ice passes through the inlet, fan, and LPC blade rows, it is continuously absorbing heat from the warmer air through the compressor blades. The amount of energy that the ice particle absorbs from the air while it experiences sublimation, melt, and evaporation will be described in the subsequent Section 4.6. The transfer of energy between the air and ice particle can be expressed as a change in enthalpy. The enthalpy increase of the ice particle is equal to the enthalpy decrease of the air, which results in a decrease of the total temperature of the air. The energy balance between the ice particle and air is determined iteratively with the results from the compressor code and the particle melt code exchanging boundary conditions of static temperature and pressure and change in enthalpy until convergence. The change in total temperature of the air through the rotor is a function of the

enthalpy rise due to the work input by the rotor, and the enthalpy that the ice particle removes from the air, as determined from the particle melt code. To account for effects of the energy exchange between the ice particle and air through the compressor rotor, the Euler equation energy balance in the compressor code has been modified to account for the enthalpy exchange between the particle and air, and is represented by the following equation, where the subscripts 1 and 2 represent the conditions at the rotor leading edge (station 1) and trailing edge (station 2), respectively (Figure 3).

$$Jc_{p_{\text{wet}}}g_c(T_2 - T_1) = (U_2C_{U2} - U_1C_{U1}) - \Delta H_{\text{melt}(2-1)} \quad (37)$$

The solution of flow conditions at the rotor exit is obtained by executing the compressor code and the particle melt code iteratively, by passing boundary conditions of static temperature and velocity from the compressor code to the MELT code, where the enthalpy change ( $\Delta H_{\text{melt}}$ ) of the ice particle is calculated, and returned to the compressor code and the change in air temperature due to the change in enthalpy is calculated. A similar iterative process is utilized to calculate the change in air temperature due to enthalpy exchange between the air and ice particles through the gaps between the rotor and stator, and through the stator vanes, where the subscripts 2 and 3 represent the inlet and outlet of the gap, and subscripts 3 and 4 represent the conditions at the stator leading and trailing edge, respectively (Figure 3). Note that when calculating the air temperature in the gap, the subscripts 3 and 4 are replaced with subscripts 2 and 3, respectively, in the following equation.

$$Jc_{p_{\text{wet}}}g_c(T_4 - T_3) = \Delta H_{\text{melt}(4-3)} \quad (38)$$

#### 4.4 Water Vapor

The COMDES code includes the capability of modeling the effects of water vapor on the gas properties of air. The model is based on a specified value of specific humidity at the inlet of the engine and compressor. The values for fluid properties of the water vapor and air are obtained in COMDES from the GASPLUS (Ref. 10) code. The mole fraction of water vapor to air is used to compute the average mole weight of the fluid ( $M_w$ ,  $M_a$ ), gas constant, and specific heat ratios for water vapor ( $c_{p_w}$ ) and air ( $c_{p_a}$ ). Molar flow rates ( $F_w$ ,  $F_a$ ) are given by

$$F_w = \frac{W_w}{M_w} \quad (39)$$

$$F_a = \frac{W_a}{M_a} \quad (40)$$

Mole fractions ( $\chi_w$ ,  $\chi_a$ ) are given by

$$\chi_w = \frac{F_w}{(F_w + F_a)} \quad (41)$$

$$\chi_a = 1 - \chi_w = \frac{F_a}{(F_w + F_a)} \quad (42)$$

Mass fractions ( $w_w$ ,  $w_a$ ) are given by

$$w_w = 1 - w_a = \frac{\chi_w M_w}{\chi_a M_a + \chi_w M_w} \quad (43)$$

$$w_a = \frac{\chi_a M_a}{\chi_a M_a + \chi_w M_w} \quad (44)$$

Average molar mass of the mixture:

$$M_{\text{wet}} = \chi_a M_a + \chi_w M_w \quad (45)$$

The specific heat of the air and water vapor mixture ( $c_{p_{\text{wet}}}$ ) is calculated with the following formula:

$$c_{p_{\text{wet}}} = w_a c_{p_a} + w_w c_{p_w} \quad (46)$$

Average gas constant of the air and water vapor mixture ( $R_{\text{wet}}$ ), in terms of the universal gas constant ( $\mathfrak{R}$ ):

$$R_{\text{wet}} = \frac{\mathfrak{R}}{(\chi_a M_a + \chi_w M_w)} = \frac{\mathfrak{R}}{M_{\text{wet}}} \quad (47)$$

The ratio of specific heats for the air and water vapor mixture  $\gamma_{\text{wet}}$ :

$$\gamma_{\text{wet}} = \frac{1}{(1 - R_{\text{wet}}/(J c_{p_{\text{wet}}}))} \quad (48)$$

## 4.5 Wet-Bulb Temperature

The  $T_{WB}$  parameter is utilized as one of the key indicators of whether there is a risk of ice accretion within one of the compressor blade rows. A mathematical derivation of a  $T_{WB}$  for application in analysis of mixed-phase icing tests has been derived and implemented into the mean-line compressor flow analysis code (Ref. 5), thus enabling the multistage code to compute the local  $T_{WB}$  at each rotor and stator leading and trailing edge and at the stator throat. In this derivation,  $T_{WB}$  is the temperature of an evaporating wet adiabatic surface and is a function of the surrounding air temperature (dry-bulb) and pressure and moisture content. The  $T_{WB}$  is obtained by equating the rate of heat transfer to a surface to the heat loss by evaporation. During this process, it is assumed that the surface remains wet and the water on the surface is at  $T_{WB}$ . It is assumed that the process is steady state, with adiabatic boundaries, except at the edge of the boundary layer. The static pressure remains constant, and the partial pressure of water vapor at the surface is assumed to be the saturation value at the surface temperature. Figure 10 illustrates the conceptual representation of the control volume for the thermodynamic balance model that is utilized to derive the  $T_{WB}$ .

An energy balance performed on the control volume  $\tilde{V}$  with the given assumptions is shown in Equation (49) where  $\rho_a$  is the density of air,  $\dot{m}_w$  is the evaporation rate,  $t$  is time,  $h$  is the convective heat transfer coefficient,  $T_M$  is the averaged control volume temperature,  $A_s$  is the surface area, and  $L_v$  is the latent heat of evaporation.

$$\rho_a \tilde{V} c_{p_a} \frac{dT_M}{dt} = h A_s (T_\infty - T_s) - \dot{m}_w L_v \quad (49)$$

Assuming a steady state process, the previous equation can be rearranged to solve for the surface temperature  $T_s$  as shown in Equation (50).

$$T_s = T_\infty - \frac{\dot{m}_w L_v}{h A_s} \quad (50)$$



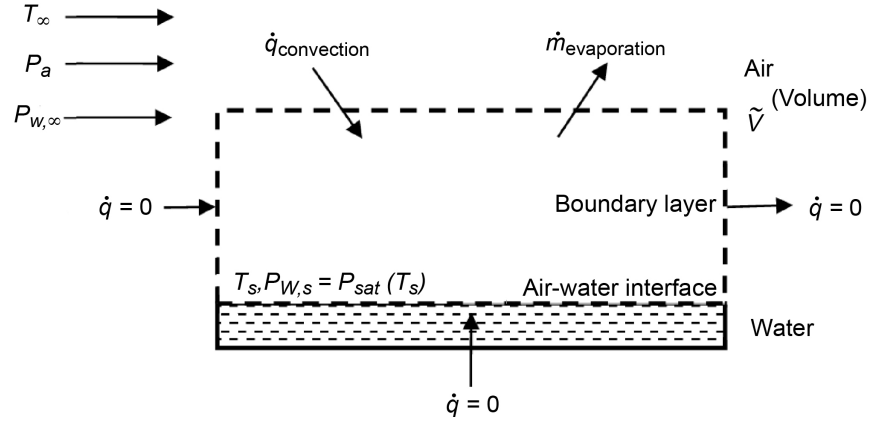


Figure 10.—Control volume for thermodynamic balance model. Temperature is  $T$ ; pressure is  $P$ ; evaporation rate of water is  $\dot{m}$ ; heat flux is  $\dot{q}$ ; control volume is  $\tilde{V}$ . Subscripts: air is  $a$ ; surface is  $s$ ; water vapor is  $w$ ; saturated is  $sat$ ; free stream is  $\infty$ .

Examination of this equation shows that  $T_{WB}$  will be less than the free-stream temperature  $T_\infty$  when there is evaporation from the surface given the assumptions previously stated.

To determine the amount of mass evaporated leaving the top of the boundary layer, it is assumed that the process is steady (i.e., no accumulation in the control volume). The evaporated water leaving the control volume is resupplied to the surface at  $T_{WB}$  through the impingement of partially melted ice crystal on the surface. It is also assumed that saturated conditions exist at the water and air interface. Under these assumptions, the mass flux leaving the control volume is shown in Equation (51) where  $h_m$  is the mass transfer coefficient.

$$\dot{m}_w = h_m A_s \rho_a (w_s - w_\infty) \quad (51)$$

Combining the mass flux leaving the control volume, Equation (51), with the energy balance, Equation (50), provides another expression for the  $T_s$ , as shown in Equation (52):

$$T_s = T_\infty - \frac{h_m}{h} L_v \rho_a (w_s - w_\infty) \quad (52)$$

Invoking the Chilton-Colburn analogy of heat and mass transfer yields another expression for the  $T_s$  equation. This analogy is valid for both local and bulk transfer coefficients as well as laminar or turbulent conditions. The properties' values are evaluated at the average of the free-stream and the surface temperature. A more detailed derivation of this expression is presented in Reference 11.

$$T_s = T_\infty - \frac{L_v}{c_{p_a} \text{Le}^{2/3}} \left( \frac{M_w}{M_a} \right) \frac{T_M}{P_a} \left[ \frac{P_{w,s}}{T_s} - \frac{P_{w,\infty}}{T_\infty} \right] \quad (53)$$

Equation (53) can be solved iteratively. The  $T_s$  is the  $T_{WB}$ , based on the prior assumptions. The dimensionless variable  $\text{Le}$  is the Lewis number.

The partial pressure of water vapor  $P_w$  at the surface is assumed to be the saturation value  $P_{sat}$  at the  $T_s$ , Equation (54). The  $P_w$  outside the boundary layer can be defined based on the ambient relative humidity,  $\phi$ , as shown in Equation (55).

$$P_{w,s} = \phi_s P_{sat,s} \quad \text{evaluated at } T_{WB} \quad (\phi_s = 1.0, \text{ fully saturated}) \quad (54)$$

$$P_{w,\infty} = \phi_\infty P_{sat,\infty} \quad \text{evaluated at } T_\infty \quad (55)$$

## 4.6 Ice Particle Melting and Evaporation

The capability to model ice particle and evaporation is available in the COMDES compressor analysis code as implemented in the subroutine named MELT. The MELT code calculates the change in phase of an ice particle which sublimates, melts, and evaporates, as it passes through the fan and LPC rotor blades and stator vanes. The ice particle residence times through the compressor of existing engines are estimated based on the velocities at the leading and trailing edge of each blade row, the blade chords, and the axial spacing of the gap between the rotors and stators. This provides a reasonable estimate of the actual residence times of the ice crystals through each blade passage and gap, resulting in an accurate estimate of the sublimation, melt, and evaporation physics. The inlet specific humidity and the ice particle concentration ( $\text{g}/\text{m}^3$ ) and size distributions are specified at the inlet to the engine. Values for ice particle size and concentrations in the atmosphere are varied parametrically, as there is limited data currently available on the ice crystal content of high-altitude convective clouds.

After the flow conditions through the blade rows have been calculated by the compressor code, the static temperatures, pressures, and velocities are passed to the ice melting subroutine to determine the rate of melting and evaporation, and thus, the local melt ratio in each blade row. The ice particle sublimation, melt, and evaporation equations are shown below. These calculations take into consideration the local temperatures, pressures, and residence times as they traverse the blade passages and gaps between blades at their mid span location. Conservation of mass on each particle results in the following equation for the ice particle size  $d$  as a function of position, where  $Sh$  is the Sherwood number,  $D_v$  is the diffusion coefficient of water vapor into air,  $x$  is the distance traveled by the particle,  $V_\infty$  is the air free stream velocity, and  $\rho_i$  is the ice particle density.

$$d^2 = d_o^2 - \frac{4Sh\rho_a D_v}{\rho_i V_\infty} (x - x_o)(w_s - w_\infty) \quad (56)$$

Similarly, the rise in specific humidity  $S_h$  is related to the amount of mass lost by the particle and is given by the following equation where  $IWC$  is the ice water content.

$$S_h = S_{h,o} + \frac{IWC}{\rho_a} \left[ 1 - \left( \frac{d}{d_o} \right)^3 \right] \quad (57)$$

Equating the heat absorbed by the particles to a decrease in enthalpy in the airstream provides the decrease in air temperature and  $L_s$  is defined as the latent heat of sublimation.

$$T_a = T_{a,o} + \frac{L_s}{c_{p_a}} \frac{IWC}{\rho_a} \left[ 1 - \left( \frac{d}{d_o} \right)^3 \right] \quad (58)$$

The enthalpy increase of the particle through the compressor is equal to the heat absorbed from conduction plus the energy provided by evaporation, resulting in the following equation for the change in ice particle temperature prior to melt. Here  $Nu$  is the Nusselt number and  $Sc$  is the Schmidt number.

$$\frac{dT_i}{dx} = \frac{\text{Nu}}{\tau_T V_\infty} (T_a - T_i) + \frac{L_s \text{Sh}}{\tau_T c_{p_a} V_\infty} \frac{\text{Pr}}{\text{Sc}} (w_\infty - w_s) \quad (59)$$

The mass fractions of water vapor in the previous equation are given by the following equations:

$$w_\infty = \phi_\infty \frac{M_w}{M_a} \frac{P_{w,\infty}}{P_a} \quad (60)$$

$$w_s = \phi_s \frac{M_w}{M_a} \frac{P_{w,s}}{P_a} \quad (\phi_s = 1.0, \text{ fully saturated}) \quad (61)$$

The nondimensionalization of the previous energy equation results in the following term for the thermal response time  $\tau_T$  where  $k$  is the thermal conductivity.

$$\tau_T = \frac{c_{p_i} \rho_i d^2}{6k_a} \quad (62)$$

The Nu and Sh numbers in these equations are provided by assuming spherical particles. If other particle geometries are of interest, the correlations can be easily modified for those cases. Prandtl number Pr is the same for all particles. The variable  $\mu$  is the dynamic viscosity.

$$\text{Pr} = \left( \frac{c_p \mu}{k} \right)_a \quad (63)$$

$$\text{Nu} = \frac{hd}{k_a} \quad (64)$$

$$\text{Nu} = 2 + 0.6 \text{Re}^{1/2} \text{Pr}^{1/3} \quad (65)$$

The Sc is defined by the following equation:

$$\text{Sc} = \frac{\mu_a}{\rho_a D_v} \quad (66)$$

The Sh is defined by the following equation:

$$\text{Sh} = \frac{h_m d}{D_v} \quad (67)$$

$$\text{Sh} = 2 + 0.6 \text{Re}^{1/2} \text{Sc}^{1/3} \quad (68)$$

The Reynolds number  $Re$  in the previous equation is based on the particle size and the relative velocity of the particle  $V_d$  with the airstream,  $V_\infty$ :

$$Re = \frac{\rho_a |V_\infty - V_d| d}{\mu_a} \quad (69)$$

Once the particle starts to melt, its temperature no longer changes. In this case, the previous energy equation can be written as a change in the enthalpy or a change in the melt fraction  $n_{\text{melt}}$  as shown in Equation (70) where  $L_f$  is the latent heat of fusion.

$$\frac{dn_{\text{melt}}}{dx} = \frac{c_{p_i}}{L_f} \frac{Nu}{\tau_T V_\infty} (T_a - T_i) + \frac{c_{p_i}}{L_f} \frac{L_s Sh}{\tau_T c_{p_a} V_\infty} \frac{Pr}{Sc} (w_\infty - w_s) \quad (70)$$

The local fluid conditions of temperature and pressure are used in the previous equations to determine the rate of sublimation, ice melt, and evaporation. The distance the ice particle traverses through the compressor rotors and stators is determined from the length of the engine inlet, and the geometry of the rotor blades, stator vanes, and the axial gaps between the rotors and stators. The distance through the fan and LPC blade rows is estimated from the true blade chord of the rotors and stators at the midspan, and the axial distance through the gaps between the blades, as illustrated in Figure 11.

Note that each stage is analyzed at the blade edges as shown in Figure 4. Here, the simple circular arc blade camber line shapes are utilized and provide a means of easily estimating the geometry of the compressor rotor blades and stator vanes. This geometry is used as the distance traveled by the water and ice particle. The COMDES and MELT codes exchange boundary conditions of pressure, temperature, and velocity to estimate the residence time of the particle through each blade row, as well as to determine the rate of ice sublimation, melt, and evaporation.

Figure 11 illustrates the ice particle path through the rotor blades in the relative frame of reference, while the path through the stator vanes is in the absolute frame of reference. The calculations of ice sublimation, melt, and evaporation are performed in increments from the leading to trailing edges of each rotor and stator, as well as through the axial gap between the blades. As the engine is flown through the flight trajectory, the ice particle sublimation, melt, and evaporation is calculated in each rotor and stator blade row. The amount of water due to sublimation and evaporation is added to the local value of specific humidity, thus having an effect on the local  $\phi$ , and likewise on the local  $T_{WB}$  calculation in the compressor code. The MELT code is also used to evaluate the particle conditions through the engine inlet, which is then provided as ice particle initial conditions at the leading edge of the fan core.

A flow chart that describes the COMDES-MELT compressor analysis is illustrated in Figure 12. The COMDES-MELT code is executed with a standard input data file that has a format that will be described in a later section. If an icing risk analysis is being conducted, each component goes through an iterative process where the ice exchanges enthalpy with the air. The stage calculation is repeated until the performance of all stages have been computed. There are several output files that result from this computation and will be described in detail in a later section of this user's manual.

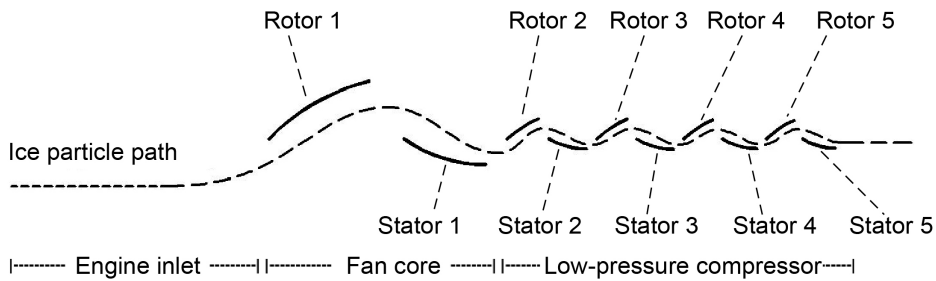


Figure 11.—Ice particle path through fan core and low-pressure compressor (LPC) stages. Rotor and stator blade mean camber line chord at the midspan section is used to estimate distance traveled and residence time of ice particles.

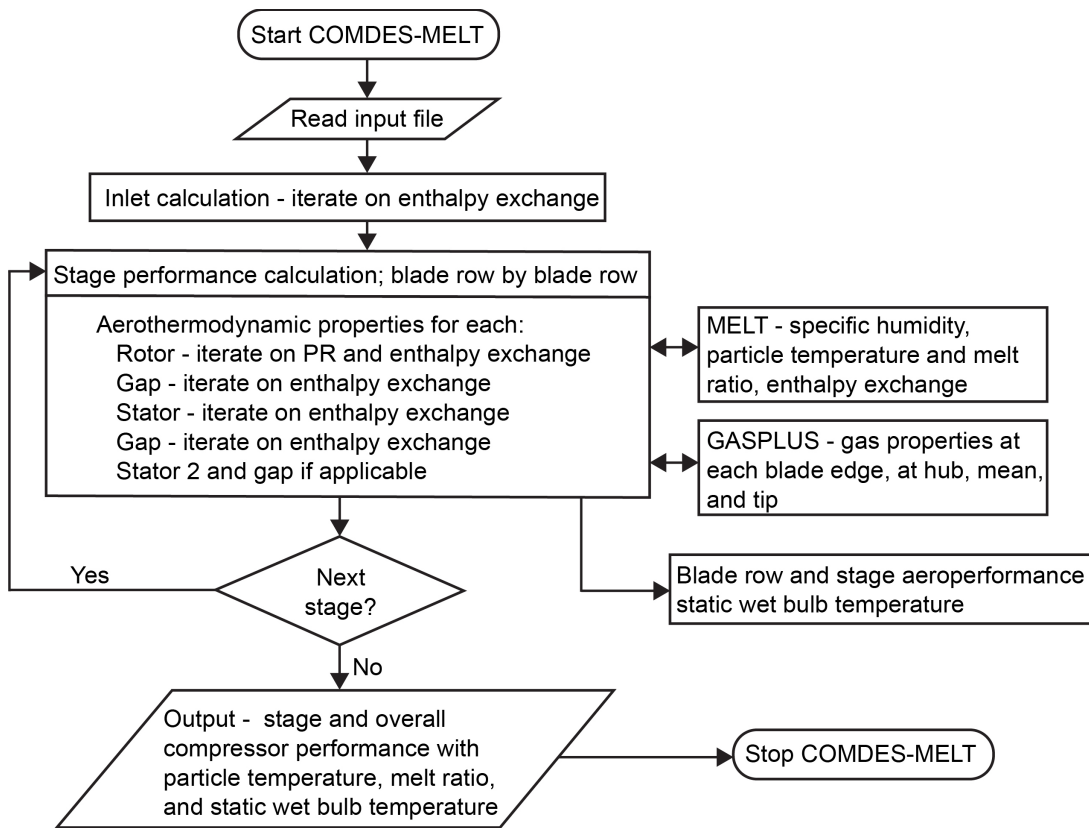


Figure 12.—Flowchart of COMDES-MELT compressor analysis code. Pressure ratio is PR.

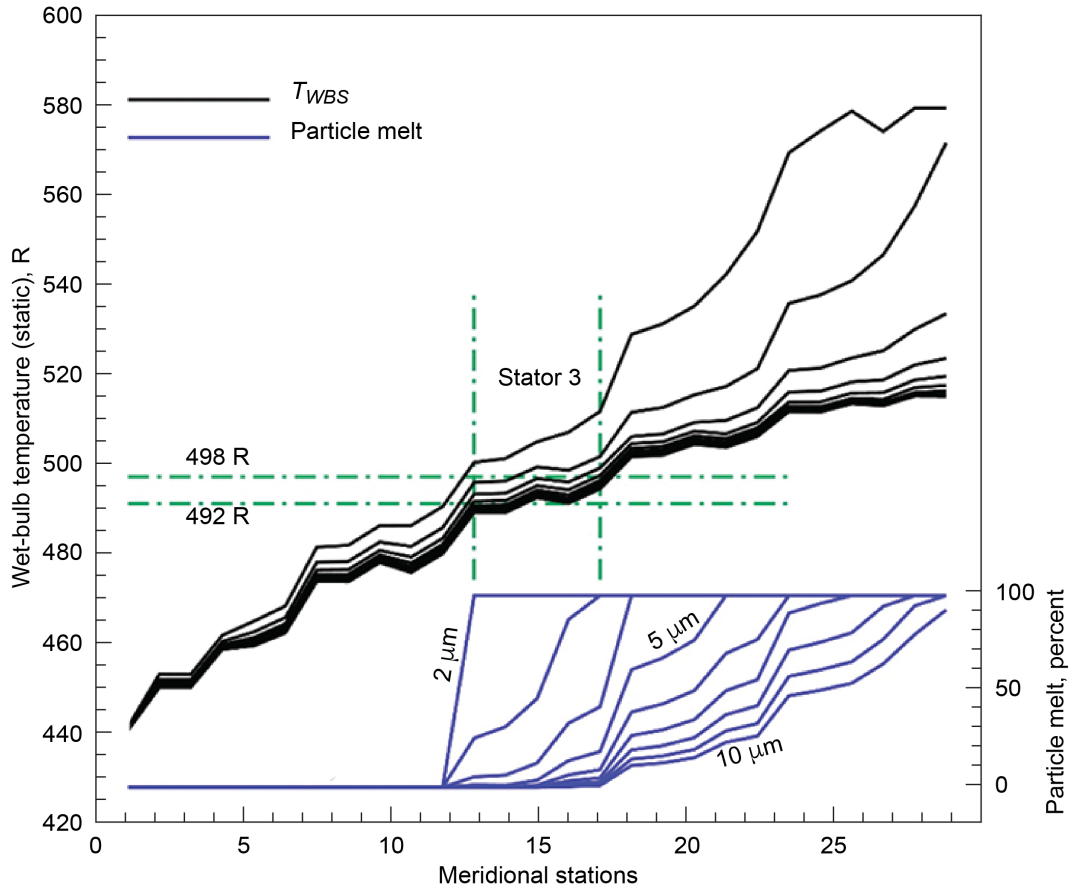


Figure 13.—Distribution of static wet-bulb temperature  $T_{WBS}$  and particle melt ratio. Risk of ice accretion indicated in stator 3 where  $T_{WBS}$  is within the range of 492 and 498 R with non-zero particle melt ratio.

The distribution of static  $T_{WB}$  and particle melt ratio for the notional high bypass ratio turbofan engine fan-core and LPC stations are illustrated in Figure 13. As the ice particle passes through the compressor components, the particle temperature increases. Sublimation, melt, and evaporation of the ice particle contributes to the change in specific humidity and temperature of the air.

For example, at an engine operating condition at an altitude of 40,000 ft with an inlet temperature 25 °R above ISA temperature, the particles have a nonzero melt ratio in stator 3 while the  $T_{WBS}$  is within the range of 492 to 498 °R, and the IWAR was 0.006. These three risk parameters indicate that there is a risk of ice accretion in stator 3 at this operating condition. The input and corresponding output files for this operating condition is provided as Example Case (1) for the ice particle size of 5  $\mu\text{m}$ . The data used to create this figure is found in the file plot.out, also included as an example for the case with a particle size of 5  $\mu\text{m}$ .

In a following section, this operating point for the notional fan and LPC is provided as an example with all the necessary input and output files used to generate the previous plot. A sensitivity study was also conducted to show the effects of prescribed slip, blockage, and efficiency values on the rotor exit conditions of temperature, flow angle, and pressure. In this case, the COMDES code was executed on the full fan rotor at its design point while varying the parameters of slip, blockage, and efficiency to understand their effects. In addition, a stage-matching example is presented, which includes computing the maps of each stage and determining the stage that limits the notional overall performance of the compressor.

The computational tool, COMDES-MELT, has been applied successfully to several turbofan engine compressor geometries that have been tested in the Propulsion Systems Laboratory at the NASA Glenn Research Center. The sole purpose of these tests and analysis was to determine if there was a risk of ice accretion in the compression system (Refs. 8 and 9) at simulated altitudes while operating in an ice crystal cloud environment. The COMDES-MELT code was used as a predictive tool to help guide the test plan to reduce the number of test points required in the expensive wind tunnel testing. The COMDES-MELT code provided engine operating conditions where there was a risk of ice accretion at specific locations within the compression system.

## 5.0 Code Installation and Execution

In this section, a general description will be provided for the installation, compilation, and execution of the code COMES-MELT along with a brief explanation of the output files that are produced.

### 5.1 Installation

The code executable, input, and output files are contained in the working directory. Most distributions of the COMDES-MELT code will be precompiled without the source. If the source code is supplied, there is a source directory where files are modified and recompiled with the procedure described as follows. This software, executable, source code, or results shall not be used for Federal Aviation Administration (FAA)/Joint Aviation Authorities (JAA) certification or airworthiness certification (i.e., 14 CFR Parts 23, 25, 27, 29, and 33).

### 5.2 Compiling

A Fortran compiler is used to compile the source codes and link the object files to produce an executable file. An example of this procedure is provided as follows:

First compiling the codes:

```
ifort -c COMDES-MELT-RV.1.f
ifort -c gasplus.f
ifort -c Icing.f90
ifort -c DropTemp.f90
```

and then linking the object files with:

```
ifort -o COMDES-MELT-RV.1.exe COMDES-MELT-RV.1.obj gasplus.obj Icing.obj DropTemp.obj
```

to produce the executable file COMDES-MELT-RV.1.exe. This executable file is then copied to a working directory.

### 5.3 Execution

Most distributions of the code will be with the executable file, COMDES-MELT-RV.1.exe, only. This code is executed in the working directory with the command:

```
COMDES-MELT-RV.1.exe
```

When the code is executed, a disclaimer statement is generated that is accepted with an input of “Y” or rejected with an input of “N.” The disclaimer statement reads: “THIS SOFTWARE, EXECUTABLE, SOURCE

CODE, OR RESULTS SHALL NOT BE USED FOR FAA/JAA CERTIFICATION OR AIRWORTHINESS CERTIFICATION (I.E., 14 CFR PARTS 23, 25, 27, 29, AND 33).” If the disclaimer is accepted the COMDES-MELT code reads the file, comdes.dat, the input file, and produces the file, comdes.out, the corresponding output file. Other relevant output files are generated and include:

- geometry.out
- Plot\_Geometry.dat
- Melt.out
- Melt\_Frac.out
- plot.out
- Stage\_Perf.out
- Comp\_Perf.out

Detail of the input file and output files data will be described in the next section followed by a section of examples. The example section includes information on how an input file is created, the sensitivity of the exit pressure, temperature, and flow angle to the slip factor, blockage, and efficiency specified as parameters. An example of stage matching that would be required if a compressor is designed using this code through analysis is also provided. The two files “Stage\_Perf.out” and “Comp\_Perf.out” are produced to give an abbreviated indication of the stage and overall compressor performance, respectively, for a given operating condition.

Two files (ERROR.LOG and fort.13) are generated during an execution of the COMDES-MELT code and are produced by the GASPLUS code when parameters are outside its valid operating range.

## 6.0 Input and Output Data Files and Variable Description

The input and output data files that are used with the COMDES-MELT code are described in this section with a brief variable explanation.

### 6.1 Introduction

In this section, the input files for executing COMDES and the resulting output files generated by COMDES will be described in detail. The dimensions and units used in the input file and the output files are outlined in Table III.

### 6.2 Units

The units used in COMDES are described in the Table III.

TABLE III.—UNITS USED IN COMDES  
INPUT AND OUTPUT FILES

Length .....	in.
Area .....	in <sup>2</sup>
Pressure.....	lbf/in <sup>2</sup>
Temperature .....	°R
Velocity.....	ft/s
Wheel speed (N1 and N2).....	RPM
Mass flow.....	lbm/s
Angle.....	degree
Drop diameter .....	µm
Ice water content .....	g/m <sup>3</sup>
Altitude .....	ft



These units are consistent between input and output files. The icing parameters are in metric units to be consistent with what is used in the airframe icing research community.

### 6.3 Input and Output File Structures

The pages that follow provide a variable listing of the input and output files followed by a full description of each variable in the order of appearance in each file. Several places in these files are redundant (i.e., stage 2, stage 3, etc.) and a note is included to show that lines were omitted for clarity.

In general, the input file includes an inlet or fan-face condition section (lines 1 to 6 below) followed by stage conditions that includes inputs of geometry, efficiency, slip factor, etc. (lines 7 to 20). If there is a tandem or second stator that input follows the stage stator (lines 21 to 25). Additional stage input data is then inserted to represent the full extent of the compressor. A region downstream of the final stage is included if a gooseneck and strut region is part of the calculation (lines 40 to 43). Sample input files are included after the input and output file descriptions, which provide guidance on the range for some of the input variables. This software, executable, source code, or results shall not be used for FAA/JAA certification, or airworthiness certification (i.e., 14 CFR Parts 23, 25, 27, 29, and 33).

#### 6.3.1 Core Program Input—List of Required Data

```

Line #
1  Title
2  Wact      Pt1      Tt1      RPM1
3  Fluid     Stages   IDUMMY*  DUMMY**
4  DropDia   InDist   RSHinlet IWC
5  Psin      DUMMY**  Tsin     DUMMY**
6  Vin       DUMMY**  Altitude IDUMMY**
STAGE Calculation variables are repeated for each Stages
7  Rltip     R1hub    R2tip    R2hub
8  Blaxial   B2axial  NBLAD    THK      CFS
9  StatorT   StatorM  StatorH  FSGV    CFMid
10 AeroB1    AeroB2   POTS     POTH    CFHub
11 EfficiT   EfficiM  EfficiH  BLEED
12 DUMMY**   DUMMY**  DUMMY**  DUMMY**
13 SlipF     Wbar24   SolidRotor RPM2
14 BetaBlade1T BetaBlade1M BetaBlade1H expo3
15 BetaBlade2T BetaBlade2M BetaBlade2H stat2
16 R3s       R3h      B3X      BetB3
17 Blan3     Blath3   Blok3    Wbar23
18 RCG       Athrt    BlokTH   Wbar2T
19 R4S       R4H      B4X      BetB4
20 Blan4     Blath4   Blok4    SolidStat1
IF 2ND STATOR←see line 15 repeat lines 16-20
21 R5s       R5h      B5X      BetB5
22 Blan5     Blath5   Blok5    Wbar46
23 RCG2     Athrt2   BlokTH2  Wbar4T
24 R6S       R6H      B6X      BetB6
25 Blan6     Blath6   Blok6    SolidStat2
STAGE 2+ Repeat lines 7-20 (or 7-25 if tandem stators) for additional stage calculations
.
. (Stage input variables are repeated for multiple stage compressors as
. describe above which is controlled by the "Stages" variable in line 3)
.
.
GooseNeck and Strut
40. R7S      R7h      B7X      Length67
41. Blan7    Blath7   Blok7    Wbar67
42. R8S      R8h      B8X      Length78
43. Blan8    Blath8   Blok8    Wbar78

```

\* IDUMMY - integer input variable that is not used in the computation.  
\*\* DUMMY - real input variable that is not used in the computation.

### 6.3.2 Core Program Input—Data Definitions (by Line Number) With Required Format

Line #

1. Title (1A100)
2. Wact Pt1 Tt1 RPM1 (4F12.4)  
 Wact—actual weight flow of fluid, lbm/s  
 Pt1—inlet total pressure, psia  
 Tt1—inlet total temperature, R  
 RPM1—physical rotational speed of the low speed shaft, RPM
3. Fluid Stages IDUMMY DUMMY (3I12,F12.4)  
 Fluid—operating fluid:  
 1—hydrogen  
 2—oxygen  
 3—nitrogen  
 4—water  
 5—air  
 6—helium  
 7—carbon dioxide  
 Stages—number of stages in the compressor; compute one stage at a time when setting up multistage  
 IDUMMY—no longer used = 0  
 DUMMY—no longer used = 0.0
4. DropDia InDist RSHinlet IWC (4F12.4)  
 DropDia—when running with ice crystals this sets the diameter of the particle,  $\mu\text{m}$  (icing risk analysis only)  
 InDist—length of the inlet, in. (icing risk analysis only)  
 RSHinlet— $\phi$  or  $S_h$  set at the inlet:  
 if  $RSHinlet > 1.0$ , it is  $\phi$ , percent  
 if  $RSHinlet \leq 1.0$ , it is  $S_h$   
 IWC — Ice water content,  $\text{g}/\text{m}^3$
5. Psin DUMMY Tsin DUMMY (4F12.4)  
 Psin—static pressure at the inlet, psia (icing risk analysis only)  
 DUMMY—no longer used = 0.0  
 Tsin—static temperature at the inlet, R (icing risk analysis only)  
 DUMMY—no longer used = 0.0
6. Vin DUMMY Altitude IDUMMY (3F12.4,I12)  
 Vin—Velocity at the inlet, ft/s (icing risk analysis only)  
 DUMMY—no longer used = 0.0  
 Altitude—altitude of operating point, ft. Note: inlet pressure determines altitude.  
 IDUMMY—no longer used = 0

STAGE 1

7. R1tip R1hub R2tip R2hub  
 R1tip—tip radius of rotor at leading edge (LE), in.  
 R1hub—hub radius of rotor at LE, in.  
 R2tip—tip radius of rotor at trailing edge (TE), in.  
 R2hub—hub radius of rotor at TE, in.

8. B1axial    B2axial    NBLAD    THK    CFS (2F12.4,I12,2I12.4)  
 B1axial—rotor lean at the LE from hub to tip, in.  
 B2axial—rotor lean at the TE from hub to tip, in.  
 NBLAD—number of rotor blades  
 THK—rotor blade thickness, in. (e.g., 0.020 in.)  
 CFS—relaxation factor used to adjust the internally calculated initial pressure ratio at the rotor tip, usually set to 1.0 but can be within the range of 0.8 to 1.1, for convergence
9. StatorT    StatorM    StatorHFSGV    CFMid (5F12.4)  
 StatorT—tip swirl from upstream stator coming into rotor LE, positive or negative angle, degree  
 StatorM—mean swirl from upstream stator coming into rotor LE, degree  
 StatorH—hub swirl from upstream stator coming into rotor LE, degree  
 FSGV—unused input, set = 1.0  
 CFMid—relaxation factor used to adjust the internally calculated initial pressure ratio at the rotor mid, usually set to 1.0 but can be within the range of 0.8 to 1.1, for convergence
10. AeroB1    AeroB2    POTS    POTH    CFHub (5F12.4)  
 AeroBL—Aeroblockage at rotor LE, 0.98 is a good first guess, 1.0 = no aeroblockage, (0.98 = 2 percent blockage)  
 AeroB2—aeroblockage at rotor TE, 0.95 is a good first guess, (0.95 ~ 5 percent blockage)  
 POTS—potential ratio = (shroud velocity)/(mid velocity) use 1.0 for axials unless wall curvature effects are known from previous computational fluid dynamics (CFD) analysis, or streamline curvature code  
 POTH—potential ratio = (hub velocity)/(mid velocity) use 1.0 for axials (see previous)  
 CFHub—relaxation factor used to adjust the internally calculated initial pressure ratio at the rotor hub, usually set to 1.0 but can be within the range of 0.8 to 1.1, for convergence
11. EfficiT    EfficiM    EfficiH    BLEED (4F12.4)  
 EfficiT—rotor efficiency at tip, adiabatic, 0.93 is a good first guess, and indicates 93 percent efficiency (this can vary from 0.87 to 0.95) to match rotor exit test  $P_t$  and  $T_t$  rake data, or two-dimensional (2D) streamline curvature results  
 EfficiM—rotor efficiency at mean, adiabatic, see EfficiT  
 EfficiH—rotor efficiency at hub, adiabatic, see EfficiT  
 BLEED—0.0 = zero bleed; 0.1 = 10 percent bleed (mass flow bleed-off from upstream stage)
12. DUMMY    DUMMY    DUMMY    DUMMY (4F12.4)  
 DUMMY—no longer used = 0.0  
 DUMMY—no longer used = 0.0  
 DUMMY—no longer used = 0.0  
 DUMMY—no longer used = 0.0
13. SlipF    Wbar24    SolidRotor    RPM2 (4F12.4)  
 SlipF—rotor slip factor, Equation (28) (0.82 to 0.87 centrifugal; 0.88 to 0.92 axial)  
 Wbar24—stator pressure loss coefficient Equation (32) (0.15 to 0.25 centrifugal radial diffusers or volute; 0.03 to 0.07 axial stators); vary to match test data (or numerical 2D streamline curvature code) wall static pressure at stator exit  
 SolidRotor—rotor solidity at tip (1.4 to 2.2 at tip)  
 RPM2—physical rotational speed of the high-pressure compressor (HPC), RPM2 needs to be set to a value less than 10.0 if LPC single spool analysis → RPM2 = RPM1
14. BetaBlade1T    BetaBlade1M    BetaBlade1H    expo3 (4F12.4)  
 BetaBlade1T—rotor blade LE tip metal angle, degree  
 BetaBlade1M—rotor blade LE mean metal angle, degree  
 BetaBlade1H—rotor blade LE hub metal angle, degree

- expo3—exponent for stator 2 incidence loss model Mach number deration. If input of expo3 = 0.0 there is no correction to loss bucket (range of 1.0 to 2.0)
15. BetaBlade2T      BetaBlade2M      BetaBlade2H      stat2 (3F12.4,I12)  
 BetaBlade2T—rotor blade TE tip metal angle, degree  
 BetaBlade2M—rotor blade TE mean metal angle, degree  
 BetaBlade2H—rotor blade TE hub metal angle, degree  
 Stat2—flag to indicate if there is a second stator (tandem): 0 = no, 2 = yes
16. R3s      R3h      B3X      BetB3 (4F12.4)  
 R3s—stator LE tip radius, from the center line, in.  
 R3h—stator LE hub radius, from the center line, in.  
 B3X—stator LE sweep (axial distance in in.)  
 BetB3—stator vane LE metal angle, at the root-mean-square (RMS) radius, degree
17. Blan3      Blath3      Blok3      Wbar23 (I12,3F12.4)  
 Blan3—number of stator vanes  
 Blath3—stator vane normal thickness, in. (e.g., 0.015 in.)  
 Blok3—stator vane LE blockage (1.0 is no aerodynamic blockage; 0.98 is 2 percent aeroblockage) vary to match test data (or numerical 2D code) wall static pressure  
 Wbar23—rotor blade to stator vane gap total pressure loss coefficient (annular duct loss between blade rows, rotor TE to stator LE) range 0.01 to 0.02
18. RCG      Athrt      BlokTH      Wbar2T (4F12.4)  
 RCG—stator vane radius of center of gravity (root-mean-square RMS radius of stator throat; utilized for centrifugal radial diffusers, where it determines the diffuser loading and the static pressure recovery from the impeller TE to the diffuser throat). For axials, it can be the RMS radius of the LE  
 Athrt—stator vane throat area at the RCG (if 0.0 code calculates an approximate value), in.<sup>2</sup>  
 BlokTH—stator vane throat blockage, (1.0 is no aerodynamic blockage; 0.98 is 2 percent aeroblockage)  
 Wbar2T—stator vane throat loss coefficient; set to 0.05
19. R4S      R4H      B4X      BetB4 (4F12.4)  
 R4s—stator TE tip radius, in.  
 R4h—stator TE hub radius, in.  
 B4X—stator TE axial distance from hub to tip, caused by sweep, in.  
 BetB4—stator vane TE metal angle, degree
20. Blan4      Blath4      Blok4      SolidStat1 (I12,3F12.4)  
 Blan4—number of stator vanes  
 Blath4—stator vane normal thickness, in.  
 Blok4—stator vane TE blockage (1.0 is no aerodynamic blockage; 0.98 is 2 percent aeroblockage) vary to match test data (or numerical 2D streamline curvature code) wall static pressure  
 SolidStat1—stator vane solidity 1.4 to 2.2
- IF 2ND STATOR ← see line 15 repeat lines 16-20
21. R5s      R5h      B5X      BetB5 (4F12.4)  
 R5s—stator LE tip radius, in.  
 R5h—stator LE hub radius, in.  
 B5X—stator LE lean, in.  
 BetB5—stator vane LE metal angle, degree
22. Blan5      Blath5      Blok5      Wbar46 (I12,3F12.4)  
 Blan5—number of stator vanes

- Blath5—stator vane normal thickness, in.  
 Blok5—stator vane LE blockage (1.0 is no aerodynamic blockage; 0.98 is 2 percent aeroblockage)  
 Wbar46—tandem stator gap total pressure loss coefficient (annular duct loss between blade rows, stator TE to stator LE) range 0.01 to 0.02
23. RCG2 Athrt2 BlokTH2 Wbar4T (4F12.4)  
 RCG2—stator vane  
 Athrt2—stator vane throat area, in<sup>2</sup>  
 BlokTH2—stator vane throat blockage (1.0 is no aerodynamic blockage; 0.98 is 2 percent aeroblockage)  
 Wbar4T—stator vane throat loss coefficient
24. R6S R6H B6X BetB6 (4F12.4)  
 R6s—stator TE tip radius, in.  
 R6h—stator TE hub radius, in.  
 B6X—stator TE lean, in.  
 BetB6—stator vane TE metal angle, degree
25. Blan6 Blath6 Blok6 SolidStat2 (I12,3F12.4)  
 Blan6—number of stator vanes  
 Blath6—stator vane normal thickness, in.  
 Blok6—stator vane TE blockage (1.0 is no aerodynamic blockage; 0.98 is 2 percent aeroblockage)  
 SolidStat2—stator 2 vane solidity

STAGE 2+ Repeat lines 7-20 (or 7-25 if tandem stators) for additional stage calculations

Gooseneck and Strut

40. R7S R7h B7X Length67 (4F12.4)  
 R7S—strut/duct tip radius (LE), in.  
 R7h—strut/duct hub radius (LE), in.  
 B7X—strut/duct lean (LE), in.  
 Length67—duct length, in.
41. Blan7 Blath7 Blok7 Wbar67 (I12,3F12.4)  
 Blan7—number of strut blades in duct; that is, 3 to 5 struts  
 Blath7—blade thickness, in.  
 Blok7—blockage in duct (1.0 is unblocked)  
 Wbar67—loss model in duct; 0.02
42. R8S R8h B8X Length78 (4F12.4)  
 R8S—strut/duct tip radius (TE), in.  
 R8h—strut/duct hub radius (TE), in.  
 B8X—strut/duct lean (TE), in.  
 Length78—strut/duct length, in.
43. Blan8 Blath8 Blok8 Wbar78 (I12,3F12.4)  
 Blan8—number of strut blades in duct; that is, 3 to 5 struts  
 Blath8—strut blade thickness, inches  
 Blok8—strut blockage (1.0 is unblocked)  
 Wbar78—strut loss model; 0.02



### 6.3.4 Core Program Output—Data Definitions (by Record Number)

Record 1:	
Title	as written in input file
Record 2:	
BLEED	bleed from upstream of stage (0.1 = 10 percent)
DPInc	incidence at the Blade RMS radius (mean)
EfDer	efficiency deration from the nominal (1.0 is no deration; 0.98 is 2 percent deration) calculated from the incidence versus loss bucket for the rotor
SH	$S_h$ at the inlet
Record 3:	
W act	actual mass flow of the gas
RPM act	actual and physical rotational speed
Pt	inlet $P_t$ at rotor 1 LE (or engine inlet, if applicable)
Tt	inlet $T_t$ at rotor 1 LE (or engine inlet, if applicable)
POTS	potential ratio at the tip and shroud (ratio of the tip and shroud to the mean meridional velocity) (1.0 to 1.2) depends on curvature, the smaller the radius of curvature, upstream of the rotor, the larger the ratio (e.g., POTS 1.2 typical for multistage centrifugal compressors)
POTH	potential ratio at the hub (ratio of the hub to the mean meridional velocity) (1.0 to 0.7) depends on curvature, the smaller the radius of curvature, upstream of the rotor, the smaller the ratio (e.g., POTH 0.8 typical for multistage centrifugal compressors)
AeroBl	aeroblockage at the LE—accounts for boundary layer blockage but does not include metal blockage
Record 4:	
W Kg/sec	actual mass flow in kg/s
Wdry	if $SH > 0$ , the actual mass flow of dry air ( $W_{dry} = W_{act}$ if $SH = 0.0$ )
H2Ovap_Lb/s	water vapor mass flow rate in the air
H2Ovap_g/m <sup>3</sup>	water vapor volumetric flow rate in the air, metric
m <sup>3</sup> /min	m <sup>3</sup> /min of air/gas flow, total (air + vapor)
IWAR	ice water flow rate to air/gas flow rate (total, air + ice/water) ratio
IWC	ice water content in g/m <sup>3</sup>
Record 5:	
W cor	corrected mass flow rate of gas, corrected with pressure and temperature only (not with mole weight)
RPM cor	corrected rotational speed
GAMMA	ratio of specific heats; $c_p/c_v$
Cp	specific heat at constant pressure
R	gas constant (1,545/gas mole weight)
Blades	number of rotor blades at the LE (if centrifugal with splitter blades, set to NBLAN in input to negative value and the number of blades at the TE will be doubled, that is, $NBLAN = -18$ then $Blades2 = 36$ )
THK	thickness normal to the camber line at the LE, diameter if the LE is circular
Area1UB	full area without metal blockage or aeroblockage, conical area (not necessarily annular)
Record 6:	
CFM	ft <sup>3</sup> /min of gas flow with vapor

SCFM	standard ft <sup>3</sup> /min at standard temperature and pressure (STP) (14.696 psia and 518.6 R)
A1/A*	area ratio of the actual annular area over the critical area (area for Mach = 1.0, choke)
Area1	total aerodynamic area reduced by the aeroblockage, conical normal to the meridional direction (washer to cone to cylinder for centrifugal with no inducer)
A*	critical area computed based on the mass flow, $P_t$ , and $T_t$ for choked flow (Mach = 1.0)
AthrRotor	calculated approximate rotor throat based on annular area and the rotor LE blade angle (beta)
ChokeMargin	amount the flow can be increased before the rotor chokes, a value of 1.8 means that the mass flow can be increased by 80 percent before the flow chokes
AnnularArea	pure annular area hub to tip no blockage, annulus face area, “washer”

Record 7:

Station 1. ROTOR LEADING EDGE CONDITIONS, STAGE 1

Record 8:

TIP, MEAN, and HUB

R1	radius from centerline to hub (input), mean (calculated), and tip (input) of the flow path
Stator	$\alpha$ from upstream stator (swirl angle and prewhirl) as measured from the axial direction
Alfa	$\alpha$
C1	absolute total flow velocity
CU1	absolute tangential flow velocity
Cm1	absolute meridional flow velocity
Mabs	absolute Mach number
Mrel	relative Mach number
U1cor	rotor tip speed corrected to STP
FlowCoeff	$C_1/U_1$ ; flow coefficient: ratio of absolute total velocity to the physical tip speed

Record 9:

BetaFlo	$\beta_F$
BetaBlade	$\beta_B$ with respect to the meridional direction (axial compressor is axial and radial compressor is radial)
Incid	incidence: difference between the flow and the blade angle, positive incidence is the norm, for high relative Mach number (>1.2) rotors; 6 is design point incidence
U1	peripheral and tangential velocity of the blade
W1	relative total gas velocity
Ps1	static pressure
Ts1	static temperature
TwetBulb1	$T_{WB}$ based on the local $T_t$ and $\phi$ , see plot.out output file for $T_{WBS}$ (static is used to determine icing risk)
RH	local value of $\phi$

Record 10:

Station 2. ROTOR EXIT (TRAILING EDGE) CONDITIONS, STAGE 1

Record 11:

B2 axial	axial projection distance on the axial line from hub to tip (always positive even in the case of negative sweep)
THK	blade thickness normal to the camber line at TE, diameter if the TE is circular
AeroBl	aerodynamic blockage, that is, 0.95 is 5 percent blockage
Blades2	number of rotor blades



Area2	total aerodynamic area reduced by the aeroblockage and blade blockage, conical normal to the meridional direction (washer to cone to cylinder for centrifugal with no inducer)
Area2UB	full area without metal blockage or aeroblockage, conical area (not necessarily annular)
Record 12:	
R2	radius from centerline to hub, mean (calculated), and tip of the flow path
C2	absolute total flow velocity
Cu2	absolute tangential flow velocity
Cm2	absolute meridional flow velocity
Ao2	local speed of sound based on static temperature
Mach2	absolute Mach number
Chord	chord length of the rotor blade
AxChord	axial chord length of the rotor blade
Rcircle	radius of curvature of the camber line
Record 13:	
U2	peripheral and tangential velocity of the blade
W2	relative total gas velocity
Wu2	relative tangential gas velocity
MachRel2	relative Mach number
DelRCu	change in tangential velocity from the inlet to the outlet at a given radius
Eff2uC	adiabatic efficiency—input at each radius; best efficiency point (BEP)
Eff2incC	efficiency after deration due to incidence loss
AvgREff	average rotor efficiency at all spans
Ws1/W2	ratio of rotor relative tip velocity to relative mean velocity used to determine the diffusion through the rotor; the “diffuser analogy” is used to determine the rotor surge point. Stable if $Ws1/W2 < 1.9$ ; surge if $Ws1/W2 > 1.9$ ; if $DiffFct > 0.62$ (see following), then that determines the surge point even if $Ws1/W2 < 1.9$
Record 14:	
Pt2	$P_t$
PR	$P_t$ ratio across the rotor (total exit to total inlet (T-T))
Ps2	static pressure
Tt2	$T_t$
TR	$T_t$ ratio across the rotor (T-T)
Ts2	static temperature
TwetBulb2	$T_{WB}$ based on $T_t$ and $\phi$
RH	$\phi$
Tt2avg	average $T_t$
PR2avg	average $P_t$ ratio (T-T)
Record 15:	
Alfa2	$\alpha$
Beta FLO	$\beta_F$
Beta BLADE	$\beta_B$ with respect to the meridional direction (axial compressor is axial and radial compressor is radial)
Deviat	deviation angle in degrees: difference between the flow angle and the blade angle
Slip F.	$\delta$ : the difference between the theoretical and absolute fluid tangential velocities normalized with the blade physical/peripheral/tangential velocity

DiffFct	$DF$ ; used to determine the diffusion through the rotor; used to determine the rotor surge point. Flow in the rotor is stable if $DiffFct < 0.62$ ; surge if $DiffFct > 0.62$ ; if $Ws1/W2$ reaches 1.9 then that determines the surge point even if $DiffFct$ is $< 0.62$
Solidity	blade solidity (see input description)
Convergence	level of convergence of pressure ratio (PR)
Iter	number of iterations to achieve convergence at each blade span, if 10,000 iterations are reached the solution did not converge, turn on the print variable (1) in the input file to get more information. Possible need to adjust the initial guess on the pressure factor or mass flow needs to be adjusted down

Record 16:

Station 3. STATOR ROW #1

Record 17:

blockage3	aerodynamic blockage, that is, 0.95 is 5 percent blockage
XBladeGap	calculated axial gap; distance between rotor TE hub to the stator LE hub
Vane#	number of stator vanes
Area3	total aerodynamic area reduced by the aeroblockage and blade blockage, conical normal to the meridional direction (washer to cone to cylinder for centrifugal)
Tt3	$T_t$
TwetBulb3	$T_{WB}$ based on $T_t$
MeltFract	ice particle melt fraction if the MELT code is used for ice particle ingestion analysis
Area3UB	full area without metal blockage or aeroblockage, conical area (not necessarily annular)

Record 18:

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE

Record 19:

R3m	RMS radius from centerline (calculated from input values of R3s and R3h)
C3	absolute total flow velocity
Cu3	absolute tangential flow velocity
Cm3	absolute meridional flow velocity
Ao3	local speed of sound based on static temperature
Mach3	local Mach number
cp 2-3	static pressure recovery coefficient from rotor TE to stator LE
Stat Ax Chd	stator axial chord

Record 20:

Pt3	$P_t$
Ps3	static pressure
Ts3	static temperature
RH3	$\phi$
FloAlpha3	$\alpha$
VaneAlpha3	$\beta_B$
Incid3	incidence angle at stator LE; difference between the $\alpha$ and the $\beta$ (design point incidence = 0 degree)

Record 21:

Station 4. STATOR / VANED DIFFUSER THROAT:

Record 22:

RCG	stator vane radius of center of gravity of the conical/annular area (RMS radius of stator throat; utilized for centrifugal radial diffusers, where it determines the diffuser loading)
-----	--

and the static pressure recovery from the impeller trailing edge to the diffuser throat).  
For axials, it can be the RMS radius of the leading edge

Cth absolute total flow velocity  
 Cuth absolute tangential flow velocity  
 Cmth absolute meridional flow velocity  
 Aoth local speed of sound based on static temperature  
 Machth local Mach number  
 cp 2-Th static pressure recovery coefficient from rotor TE to stator throat. Note: this parameter has been correlated with stall for centrifugal compressor stages having radial vaned diffusers. Stall is likely to occur if the value is  $> 0.45$   
 Stat Chord stator chord; see file geometry.out

Record 23:

BlockageTh aerodynamic blockage, that is 0.95 is 5 percent blockage  
 PtTh  $P_t$  at the stator throat  
 PsTh static pressure at the stator throat  
 TsTh static temperature at the stator throat  
 TwetBulbTh  $T_{WB}$  based on the  $T_t$  at the stator throat  
 AreaTh aerodynamic throat area  
 w2-Th loss coefficient (input) rotor exit to throat of stator (or radial diffuser throat)  
 DiffFact4  $DF$ ; stable if DiffFact4  $< 0.5$  for axial stators

Record 24:

Station 5. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):

Record 25:

R4 RMS radius from centerline (calculated from input values of R4s and R4h)  
 C4 absolute total flow velocity  
 Cu4 absolute tangential flow velocity  
 Cm4 absolute meridional flow velocity  
 Ao4 local speed of sound based on static temperature  
 Mach4 Mach number  
 cp 3-4 static pressure recovery coefficient from stator LE to stator TE. Note: stator stall can occur above values of 0.5 ( $< 0.7$  for radial diffuser of centrifugal stage)  
 Stator Gap axial gap distance between stator TE to the LE of the next blade row along the hub

Record 26:

Blockage4 aerodynamic blockage, that is, 0.95 is 5 percent blockage  
 Ps4 static pressure  
 Ts4 static temperature  
 TwetBulb4  $T_{WB}$  based on  $T_t$   
 VaneAlpha4 stator TE physical/metal vane angle  
 Vane Thk4 stator thickness normal to the camber line at the TE  
 w2-4OD off-design pressure loss coefficient (calculated from input w2-4 derated by the incidence angle)  
 cp 2-4 static pressure recovery coefficient from rotor TE to stator TE

Record 27:

STAGE EXIT CONDITIONS, STAGE 1

Record 28:

Eff4 stage efficiency

Pt4	$P_t$ at stage exit
PR	stage pressure ratio (total to total)
TR	stage temperature ratio (total to total)
Tt4	$T_t$ at stage exit
Del T	$T_t$ rise through the rotor
Ns	specific speed; for axials between 200 and 500; for centrifugals between 40 and 120
Ns nondim	specific speed, nondimensional
Record 29:	
Del Enthalpy	change in enthalpy for the stage
Del_H/U <sup>2</sup>	head coefficient
GHP	gas horsepower
Reynolds#	Reynolds number based on blade span and mean absolute velocity at the rotor LE
Specif Humid	local $S_h$
Area4	total aerodynamic area reduced by the aeroblockage and blade blockage, conical normal to the meridional direction (washer to cone to cylinder for centrifugal)
Area4UB	full area without metal blockage or aeroblockage, conical area (not necessarily annular)
Record 30:	
Station 1. ROTOR LEADING EDGE CONDITIONS, STAGE 2 – N (N is the total number of stages)	
.	
Stage 2 – N (N is the total number of stages) output repeat of variables stations 1 - 5 (records 1 - 29) from previous information	
.	
.	
.	
Record 31:	
OVERALL EXIT CONDITIONS; ALL N STAGES (N is the total number of stages)	
Record 32:	
Del Enthalpy	overall change in enthalpy for N stage compressor
GHP	overall gas horsepower for N stage compressor (working fluid not including horsepower due to bearing and seal friction losses)
MassFloS1cor	corrected mass flow into stage 1 same as Wcor
OPR	overall pressure ratio for N stages (T-T)
Efficiency	overall efficiency for N stages (T-T)
RotorInc	rotor incidence of stage 1 at RMS radius
TR	overall temperature ratio for N stages (T-T)
AxHubLen	total axial length for an axial compressor including gaps; measured along the hub

## 7.0 Input File Setup and Example Cases for Axial, Centrifugal, and Mixed Axial and Centrifugal

This section describes how an input file is constructed to match design conditions and offdesign analysis. Adjustments to the slip factor, efficiency, and blockage models are necessary to match the compressor data, on and off design, see Section 7.1.5. The details for these example cases include the input data file followed by the corresponding output files including the files “geometry.out” and “Plot\_Geometry.dat.” The output files “plot.out” and “Melt\_Frac.out” will be described in the section where an icing risk analysis was conducted.

## 7.1 Example Case (1): 40,000 lbf Thrust Turbofan Bypass Ratio = 5.0

The input and output files that are used and generated for the full fan of this example case are provided along with a plot of the stage blade geometries.

### 7.1.1 Full Fan

See Figure 14 for a plot of the stage blade geometries.

#### 7.1.1.1 Input File

comdes.dat

```

Fan-FULL Bypass Ratio = 5.0 Design Point
1060.00 15.107 522.8 3776.0 Wact Pt1 Tt1 RPM
5 1 0 0.0 Fluid Stages IDUMMY DUMMY
0.00 8.000 0.00 0.0 DropDia InDist RHinlet IWC ***icing***
14.30 0.0 536.0 0.0 Psin DUMMY Tsin DUMMY ***icing***
318.0 0.0 1000. 0 Vin DUMMY Altitude IDUMMY ***icing***
39.25 12.5100 37.5 15.00 1 R1tip Rlhub R2tip R2hub
0.7000 0.700 32 0.050 1.0 Blaxial B2axial NBLAD THK CFS
0.0 0.0000 0.000 1.000 0.9 StatorT StatorM StatorH FSGV CFMid
0.9800 0.9500 1.0000 1.000 0.9 AeroB1 AeroB2 POTS POTH CFHub
0.915 0.915 0.915 0.0000 EfficiT EfficiM EfficiH BLEED
0.000 0.000 0.000 0.000 DUMMY DUMMY DUMMY DUMMY
0.9250 0.035 1.66 0.00 SlipF DiffLoss Solidity RPM2
60.00 54.2 38.62 2.0 BetaBlade1T BetaBlade1M BetaBlade1H expo3
56.5 42.50 -9.3 0 BetaBlade2T BetaBlade2M BetaBlade2H stat2
37.5 15.0 0.0 36.0 R3s R3h B3X BetB3
33 0.06 0.95 0.01 Blan3 Blath3 Blok3 Wbar23
33.0 0.000 0.95 0.015 RCG Athrt BlokTH Wbar2T
37.5 15.0 0.0 0.0 R4S R4H B4X BetB4
33 0.06 0.95 1.40 Blan4 Blath4 Blok4 SolidStat
37.5 15.0 0.00 2.0 R7S R7h B7X Length67
4 0.01 0.98 0.000 Blan7 Blath7 Blok7 Wbar67
37.5 15.0 0.0 2.0 R8S R8h B8X Length78
4 0.01 0.98 0.00 Blan8 Blath8 Blok8 Wbar78

```

#### 7.1.1.2 Output Files

comdes.out

```

*****
***** AXIAL & CENTRIFUGAL COMPRESSOR BLADE DESIGN 1-D *****
***** COMDES-MELT Release Version 1 *****
***** Gasplus Fluid Properties *****
*****
Fan-FULL Bypass Ratio = 5.0 Design Point

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 1
BLEED = 0.000 DPInC = 7.807 EfDer = 0.999 SH = 0.000000E+00

W act RPM act Pt Tt POTS POTH AeroB1
1060.0000 3776.0000 15.1069 522.8000 1.0000 1.0000 0.9800

W Kg/sec Wdry H2Ovap_Lb/s H2Ovap_g/m^3 m^3/min IWAR IWC
481.8182 1060.0000 0.0000 0.0000 25665.4922 0.0000 0.0000

W cor RPM cor GAMMA Cp R Blades THK ArealUB
1035.5107 3761.1646 1.4014 0.2486 53.3483 32 0.0500 4349.6499

CFM SCFM Al/A* Areal A* AthrRotor ChokeMargin AnnularArea
906368.250 831728.375 1.4133 4262.6572 3016.0134 2749.8923 0.9118 4348.1602

Station 1. ROTOR LEADING EDGE CONDITIONS, STAGE 1
R1 Stator Alfa Cl CU1 Cml Mabs Mrel Ulcor FlowCoeff
TIP 39.25 0.00 -0.02 510.31 -0.18 510.31 0.46 1.27 1288.24 0.395
MEAN 29.13 0.00 -0.02 510.31 -0.18 510.31 0.46 0.99
HUB 12.51 0.00 -0.02 510.31 -0.18 510.31 0.46 0.60

```

	BetaFlo	BetaBlade	Incid	U1	W1	Ps1	Ts1	TwetBulb1	RH	
TIP	68.47	60.00	8.47	1293.36	1390.56	13.03	501.07	498.16	0.00	
MEAN	62.01	54.20	7.81	959.87	1087.25	13.03	501.07	498.16	0.00	
HUB	38.94	38.62	0.32	412.23	656.12	13.03	501.07	498.16	0.00	
Station	2. ROTOR EXIT CONDITIONS, STAGE 1 SOLUTION IS CONVERGED									
B2 axial	THK	AeroBl	Blades2	Area2	Area2UB					
	0.7000	0.0500	0.9500	32	3492.9453	3712.8020				
	R2	C2	Cu2	Cm2	Ao2	Mach2	Chord	AxChord	Rcircle	
TIP	37.50	643.97	455.96	454.75	1179.92	0.55	12.22	7.24	200.12	
MEAN	28.56	651.14	398.93	514.62	1147.89	0.57	10.75	7.55	52.75	
HUB	15.00	825.89	557.06	609.74	1107.75	0.75	8.31	8.04	10.23	
	U2	W2	Wu2	MachRel2	DelRCu	Eff2uC	Eff2incC	AvgREff	Ws1/W2	
TIP	1235.69	902.66	779.73	0.77	17105.38	0.92	0.91			
MEAN	941.08	747.50	542.14	0.65	11398.34	0.92	0.91	0.91	1.84	
HUB	494.28	612.96	62.78	0.55	8358.12	0.92	0.91			
	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
TIP	25.23	1.67	20.60	613.37	1.17	578.77	542.20	0.00		
MEAN	21.44	1.42	17.23	583.15	1.12	547.77	529.12	0.00	587.86	1.46
HUB	19.60	1.30	13.54	567.05	1.08	510.14	521.59	0.00		
	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter	
TIP	45.08	59.75	56.50	3.25	0.93	0.45	1.66	0.00009	71	
MEAN	37.78	46.49	42.50	3.99	0.93	0.41	1.88	0.00014	48	
HUB	42.41	-5.88	-9.30	3.42	0.93	0.20	3.38	0.00059	34	
Station	3. STATOR ROW #1									
blockage3	XBladeGap	Vane#	Area3	Tt3	TwetBulb3	MeltFract	Area3UB			
	0.9500	2.0099	33	3483.1338	587.8585	531.2699	0.0000	3711.0066		
	ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE									
R3m	C3	Cu3	Cm3	Ao3	Mach3	cp 2-3	Stat Ax Chd			
	28.5591	642.1721	398.9342	503.2261	1153.8083	0.5566	0.1255	9.5067		
Pt3	Ps3	Ts3	RH3	FloAlpha3	VaneAlpha3	Incid3				
	22.0406	17.8555	553.4500	0.0000	38.4057	36.0000	2.4057			
Station	4. STATOR / VANED DIFFUSER									
RCG	Cth	Cuth	Cmth	Aoth	Machth	cp 2-Th	Stat Chord			
	33.0000	521.3896	129.6643	505.0092	1165.8477	0.4472	0.4086	9.9960		
BlockageTh	PtTh	PsTh	TsTh	TwetBulbTh	AreaTh	w2-Th	DiffFact4			
	0.9500	21.9760	19.1552	565.1891	531.2116	3477.8428	0.0230	0.4629		
Station	5. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):									
R4	C4	Cu4	Cm4	Ao4	Mach4	cp 3-4	Stator Gap			
	28.5591	487.3962	0.0000	487.3962	1168.7524	0.4170	0.3756	2.3767		
Blockage4	Ps4	Ts4	TwetBulb4	VaneAlpha4	Vane Thk4	w2-4OD	cp 2-4			
	0.9500	19.4273	568.0524	531.1416	0.0000	0.0600	0.0386	0.4634		
	STAGE EXIT CONDITIONS, STAGE 1									
Eff4	Pt4	PR	TR	Tt4	Del T	Ns	Ns nondim			
	0.9009	21.8987	1.4496	1.1244	587.8585	65.0585	390.5479	3.0275		
Del Enthalpy	Del_H/U^2	GHP	Reynolds#	Specif Humid	Area4	Area4UB				
	404965.062	0.265	24258.039	7066593.5	0.00000	3392.49585	3711.00635			
	OVERALL EXIT CONDITIONS; ALL 1 STAGES									
Del Enthal	DelHT/U1^2	GHP	MassFloS1cor	OPR	Efficiency	Rotor1Inc	TR	AxHubLen		
	404965.06	0.2421	24258.0391	1035.511	1.4496	0.90088	7.8072	1.1244	21.933	

geometry.out—Key geometric details for a compressor stage.

	R1	Beta1	R2	Beta2	Sol	Rc	Chord	AxC	X1	Y1	X2	Y2	Theta
TIP	39.250	60.000	37.500	56.500	1.660	200.120	12.223	6.432	173.309	100.060	166.878	110.454	3.500
MID	29.130	54.200	28.559	42.500	1.880	52.751	10.753	7.146	42.785	30.857	35.638	38.892	11.700
HUB	12.510	38.620	15.000	-9.300	3.383	10.232	8.310	8.040	6.386	7.994	1.654	10.097	47.920

Blade > Stator Gap (2->3) 2.009935													
	R3	Beta3	R4	Beta4	Sol	Rc	Chord	AxC	X3	Y3	X4	Y4	Theta
TIP	37.500	36.000	37.500	0.000	1.400	16.174	9.996	9.507	9.507	13.085	0.000	16.174	36.000
MID	28.559	36.000	28.559	0.000	1.783	16.174	9.996	9.507	9.507	13.085	0.000	16.174	36.000
HUB	15.000	36.000	15.000	0.000	3.394	16.174	9.996	9.507	9.507	13.085	0.000	16.174	36.000

Stator > Blade Gap (4->1) 2.376685			
Cumul Axial Length (1->5)	Stage	21.93310	
		1	

Data geometry description at each rotor, gap, stator, see Figure 3 and Figure 4:

R1 radius from centerline to hub, mid, and tip at the rotor LE  
 Beta1 rotor LE hub, mid, and tip metal angle, degree  
 R2 radius from centerline to hub, mid, and tip at the rotor TE  
 Beta2 rotor TE hub, mid, and tip metal angle, degree  
 Sol solidity at hub, mid, and tip of rotor  
 Rc arc radius at the rotor midchord  
 Chord chord of the rotor at the hub, mid, and tip  
 AxC axial chord of the rotor at the hub, mid, and tip  
 X1 axial distance from blade LE to circular arc center coordinates, rotor  
 Y1 tangential distance from blade LE to circular arc center coordinates, rotor  
 X2 axial distance from blade TE to circular arc center coordinates, rotor  
 Y2 tangential distance from blade TE to circular arc center coordinates, rotor  
 Theta included angle between LE and TE of the rotor

Blade > Stator Gap (2->3) axial gap distance between rotor and stator

R3 radius from centerline to hub, mid, and tip at the stator LE  
 Beta3 stator LE hub, mid, and tip metal angle, degree  
 R4 radius from centerline to hub, mid, and tip at the stator TE  
 Beta4 stator TE hub, mid, and tip metal angle, degree  
 Sol solidity at hub, mid, and tip of stator  
 Rc arc radius at the stator midchord  
 Chord chord of the stator at the hub, mid, and tip  
 AxC axial chord of the stator at the hub, mid, and tip  
 X3 axial distance from blade LE to circular arc center coordinates equivalent to X<sub>1</sub> in Figure 4, stator  
 Y3 tangential distance from blade LE to circular arc center coordinates equivalent to Y<sub>1</sub> in Figure 4, stator  
 X4 axial distance from blade TE to circular arc center coordinates equivalent to X<sub>2</sub> in Figure 4, stator  
 Y4 tangential distance from blade TE to circular arc center coordinates equivalent to Y<sub>2</sub> in Figure 4, stator  
 Theta included angle between LE and TE of the stator

Stator > Blade Gap (4->1) axial gap distance between stator and following rotor

Cumul Axial Length (1->5) Stage axial length for complete stage at the hub

Plot\_Geometry.dat—Rotor blade and stator vane geometries for plotting at the tip, mid, and hub circular arc camber line sections.

```

TITLE = "Blade Geometry"
VARIABLES = i Thetas xs ys Thetam xm ym Thetah xh yh
ZONE T =" Blade 1", I = 41.0
  1  30.000 -3.216 -5.374 35.800 -3.573 -4.430 51.380 -4.020 -1.960
  2  30.087 -3.063 -5.110 36.092 -3.415 -4.212 52.578 -3.851 -1.829
  3  30.175 -2.909 -4.845 36.385 -3.256 -3.994 53.776 -3.680 -1.700
  4  30.262 -2.756 -4.581 36.677 -3.096 -3.778 54.974 -3.506 -1.576
  5  30.350 -2.601 -4.317 36.970 -2.934 -3.562 56.172 -3.330 -1.455
  6  30.437 -2.447 -4.054 37.262 -2.772 -3.348 57.370 -3.151 -1.337
  7  30.525 -2.292 -3.790 37.555 -2.608 -3.134 58.568 -2.969 -1.224
  8  30.612 -2.136 -3.527 37.847 -2.443 -2.921 59.766 -2.786 -1.114
  9  30.700 -1.981 -3.264 38.140 -2.278 -2.708 60.964 -2.600 -1.009
 10  30.787 -1.824 -3.002 38.432 -2.111 -2.497 62.162 -2.412 -0.907
 11  30.875 -1.668 -2.739 38.725 -1.943 -2.287 63.360 -2.221 -0.809
 12  30.962 -1.511 -2.477 39.017 -1.774 -2.077 64.558 -2.029 -0.715
 13  31.050 -1.353 -2.215 39.310 -1.604 -1.868 65.756 -1.835 -0.625
 14  31.137 -1.195 -1.953 39.602 -1.433 -1.660 66.954 -1.639 -0.539
 15  31.225 -1.037 -1.692 39.895 -1.260 -1.453 68.152 -1.441 -0.458
 16  31.312 -0.879 -1.431 40.187 -1.087 -1.247 69.350 -1.242 -0.380
 17  31.400 -0.719 -1.170 40.480 -0.913 -1.042 70.548 -1.041 -0.307
 18  31.487 -0.560 -0.909 40.772 -0.738 -0.837 71.746 -0.839 -0.237
 19  31.575 -0.400 -0.648 41.065 -0.561 -0.634 72.944 -0.635 -0.173
 20  31.662 -0.240 -0.388 41.357 -0.384 -0.431 74.142 -0.430 -0.112
 21  31.750 -0.079 -0.128 41.650 -0.205 -0.229 75.340 -0.223 -0.056
 22  31.837 0.082 0.132 41.942 -0.026 -0.029 76.538 -0.016 -0.004
 23  31.925 0.243 0.391 42.235 0.155 0.171 77.736 0.193 0.044
 24  32.012 0.405 0.650 42.527 0.336 0.370 78.934 0.402 0.087
 25  32.100 0.567 0.909 42.820 0.519 0.568 80.132 0.613 0.126
 26  32.187 0.730 1.168 43.112 0.702 0.765 81.330 0.824 0.160
 27  32.275 0.893 1.427 43.405 0.887 0.961 82.528 1.036 0.191
 28  32.362 1.056 1.685 43.697 1.072 1.156 83.726 1.248 0.216
 29  32.450 1.220 1.943 43.990 1.259 1.351 84.924 1.461 0.237
 30  32.538 1.384 2.201 44.282 1.446 1.544 86.122 1.674 0.254
 31  32.625 1.549 2.458 44.575 1.635 1.736 87.320 1.888 0.266
 32  32.713 1.714 2.716 44.867 1.825 1.928 88.518 2.102 0.274
 33  32.800 1.879 2.973 45.160 2.015 2.118 89.716 2.316 0.277
 34  32.888 2.045 3.229 45.452 2.206 2.307 90.914 2.530 0.276
 35  32.975 2.211 3.486 45.745 2.399 2.496 92.112 2.743 0.270
 36  33.063 2.377 3.742 46.037 2.592 2.683 93.310 2.957 0.260
 37  33.150 2.544 3.998 46.330 2.786 2.870 94.508 3.171 0.246
 38  33.238 2.712 4.254 46.622 2.982 3.055 95.706 3.384 0.227
 39  33.325 2.879 4.509 46.915 3.178 3.240 96.904 3.596 0.203
 40  33.413 3.047 4.765 47.207 3.375 3.423 98.102 3.808 0.175
 41  33.500 3.216 5.020 47.500 3.573 3.606 99.300 4.020 0.143
ZONE T =" Stator 1", I = 41.0
  1  54.000 5.296 2.375 54.000 5.296 2.375 54.000 5.296 2.375
  2  54.900 5.503 2.227 54.900 5.503 2.227 54.900 5.503 2.227
  3  55.800 5.712 2.083 55.800 5.712 2.083 55.800 5.712 2.083
  4  56.700 5.923 1.941 56.700 5.923 1.941 56.700 5.923 1.941
  5  57.600 6.137 1.804 57.600 6.137 1.804 57.600 6.137 1.804
  6  58.500 6.352 1.669 58.500 6.352 1.669 58.500 6.352 1.669
  7  59.400 6.570 1.538 59.400 6.570 1.538 59.400 6.570 1.538
  8  60.300 6.790 1.410 60.300 6.790 1.410 60.300 6.790 1.410
  9  61.200 7.011 1.286 61.200 7.011 1.286 61.200 7.011 1.286
 10  62.100 7.235 1.166 62.100 7.235 1.166 62.100 7.235 1.166
 11  63.000 7.460 1.049 63.000 7.460 1.049 63.000 7.460 1.049
 12  63.900 7.688 0.935 63.900 7.688 0.935 63.900 7.688 0.935
 13  64.800 7.917 0.825 64.800 7.917 0.825 64.800 7.917 0.825
 14  65.700 8.147 0.719 65.700 8.147 0.719 65.700 8.147 0.719
 15  66.600 8.380 0.616 66.600 8.380 0.616 66.600 8.380 0.616
 16  67.500 8.614 0.517 67.500 8.614 0.517 67.500 8.614 0.517
 17  68.400 8.849 0.422 68.400 8.849 0.422 68.400 8.849 0.422
 18  69.300 9.086 0.330 69.300 9.086 0.330 69.300 9.086 0.330
 19  70.200 9.324 0.242 70.200 9.324 0.242 70.200 9.324 0.242
 20  71.100 9.564 0.158 71.100 9.564 0.158 71.100 9.564 0.158
 21  72.000 9.805 0.077 72.000 9.805 0.077 72.000 9.805 0.077
 22  72.900 10.047 0.001 72.900 10.047 0.001 72.900 10.047 0.001
 23  73.800 10.291 -0.072 73.800 10.291 -0.072 73.800 10.291 -0.072
 24  74.700 10.535 -0.141 74.700 10.535 -0.141 74.700 10.535 -0.141
 25  75.600 10.781 -0.206 75.600 10.781 -0.206 75.600 10.781 -0.206

```



26	76.500	11.027	-0.267	76.500	11.027	-0.267	76.500	11.027	-0.267
27	77.400	11.275	-0.325	77.400	11.275	-0.325	77.400	11.275	-0.325
28	78.300	11.523	-0.378	78.300	11.523	-0.378	78.300	11.523	-0.378
29	79.200	11.772	-0.428	79.200	11.772	-0.428	79.200	11.772	-0.428
30	80.100	12.022	-0.473	80.100	12.022	-0.473	80.100	12.022	-0.473
31	81.000	12.273	-0.515	81.000	12.273	-0.515	81.000	12.273	-0.515
32	81.900	12.524	-0.553	81.900	12.524	-0.553	81.900	12.524	-0.553
33	82.800	12.776	-0.587	82.800	12.776	-0.587	82.800	12.776	-0.587
34	83.700	13.028	-0.617	83.700	13.028	-0.617	83.700	13.028	-0.617
35	84.600	13.281	-0.642	84.600	13.281	-0.642	84.600	13.281	-0.642
36	85.500	13.534	-0.664	85.500	13.534	-0.664	85.500	13.534	-0.664
37	86.400	13.787	-0.682	86.400	13.787	-0.682	86.400	13.787	-0.682
38	87.300	14.041	-0.696	87.300	14.041	-0.696	87.300	14.041	-0.696
39	88.200	14.295	-0.706	88.200	14.295	-0.706	88.200	14.295	-0.706
40	89.100	14.549	-0.712	89.100	14.549	-0.712	89.100	14.549	-0.712
41	90.000	14.803	-0.714	90.000	14.803	-0.714	90.000	14.803	-0.714

Data geometry description for the hub, mid, and tip circular arcs for the rotor blade and stator vane of each compressor stage, see Figure 14.

The data for this example case consists of a single stage: a rotor blade and stator vane. If a larger compressor was considered, the stage geometric data would be repeated to be consistent with the number of stages analyzed. The first and second lines in the file contain the title and the list of variables, respectively, that are available for plotting. There are a total of 41 points that represent each circular arc airfoil camber line. Each “ZONE” represents a rotor blade or a stator vane at the tip ( $\theta_s, x_s, y_s$ ), mid ( $\theta_m, x_m, y_m$ ), and hub ( $\theta_h, x_h, y_h$ ) section. The variables  $\theta, x,$  and  $y$  are the triples that are the angle and  $x, y$  coordinates of the circular arc sweep through the 41 point discretization of the circular arc camber line from the LE to TE of the blade or vane section. Figure 14 illustrates a plot of the data above.

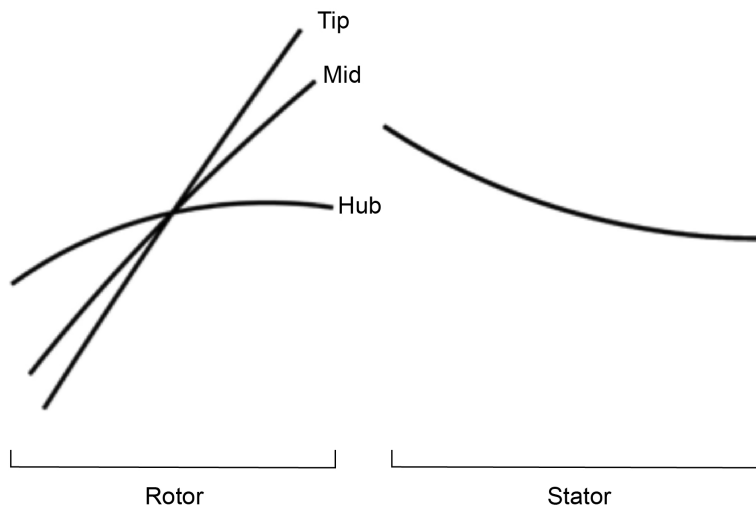


Figure 14.—Rotor blade and stator vane circular arc camber line data plotted from “Plot\_Geometry.dat” file. Single stage is plotted showing rotor blade (hub, mid, and tip) and stator vane (mid) circular arc camber line geometries.

## 7.1.2 Fan Bypass

The input and output files in this section are for the analysis of the fan bypass for this example case.

### 7.1.2.1 Input File

comdes.dat

```

Fan-BYPASS   Bypass Ratio = 5.0   Design Point
883.3        15.107        522.8        3776.0
5            1            0            0.0
0.00        8.000        0.00        0.0
14.30       0.0        536.0        0.0
318.0       0.0        1000.        0
39.25       20.6300       37.5        20.63   1 Rltip
0.7000      0.700        32          0.050   1.0
0.0         0.0000       0.000       1.000   0.9
0.9800     0.9500       1.0000      1.000   0.9
0.915      0.915        0.915       0.0000  EfficiT  EfficiM  EfficiH  BLEED
0.000      0.000        0.000       0.0000  DUMMY    DUMMY    DUMMY    DUMMY
0.9250     0.035        1.66        0.00    SlipF    DiffLoss  Solidity  RPM2
60.00      54.2         50.47       2.0     BetaBlade1T BetaBlade1M BetaBlade1H expo3
56.5       44.00        24.2        0       BetaBlade2T BetaBlade2M BetaBlade2H stat2
37.5       20.63        0.0         40.0    R3s      R3h      B3X      BetB3
33         0.06        0.95        0.01    Blan3    Blath3   Blok3    Wbar23
0.0        0.0000      0.95        0.015   RCG      Athrt    BlokTH   Wbar2T
37.5       20.63        0.0         0.0     R4S      R4H      B4X      BetB4
33         0.06        0.95        1.40    Blan4    Blath4   Blok4    SolidStat
37.5       20.63        0.00        2.0     R7S      R7h      B7X      Length67
4          0.01        0.98        0.000   Blan7    Blath7   Blok7    Wbar67
37.5       20.63        0.0         2.0     R8S      R8h      B8X      Length78
4          0.01        0.98        0.00    Blan8    Blath8   Blok8    Wbar78

```

### 7.1.2.2 Output Files

comdes.out

```

*****
***** AXIAL & CENTRIFUGAL COMPRESSOR BLADE DESIGN 1-D *****
***** COMDES-MELT Release Version 1 *****
***** Gasplus Fluid Properties *****
*****
Fan-BYPASS   Bypass Ratio = 5.0   Design Point

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 1
BLEED = 0.000   DPinc = 8.521   EfDer = 0.997   SH = 0.000000E+00

W act      RPM act      Pt      Tt      POTS      POTH      AeroB1
883.3000   3776.0000   15.1069  522.8000  1.0000    1.0000    0.9800

W Kg/sec   Wdry      H2Ovap_Lb/s  H2Ovap_g/m^3  m^3/min   IWAR      IWC
401.5000   883.3000  0.0000     0.0000     21597.3047  0.0000    0.0000

W cor      RPM cor      GAMMA     Cp      R      Blades    THK      ArealUB
862.8931   3761.1646   1.4014    0.2486   53.3483   32        0.0500   3505.2422

CFM        SCFM        A1/A*     Areal     A*      AthrRotor  ChokeMargin  AnnularArea
762701.562  693080.750  1.3668    3435.1372  2513.2429  2220.7261  0.8836   3502.7678

Station 1. ROTOR LEADING EDGE CONDITIONS, STAGE 1

      Rl      Stator  Alfa     Cl      CU1     Cm1     Mabs     Mrel     Ulcor     FlowCoeff
TIP   39.25    0.00    -0.02   532.87  -0.18   532.87   0.49    1.28  1288.24  0.412
MEAN  31.35    0.00    -0.02   532.87  -0.18   532.87   0.49    1.06
HUB   20.63    0.00    -0.02   532.87  -0.18   532.87   0.49    0.79

      BetaFlo  BetaBlade  Incid    U1      W1      Ps1     Ts1     TwetBulb1  RH
TIP   67.61    60.00     7.61    1293.36  1399.00  12.85   499.11   498.16     0.00
MEAN  62.72    54.20     8.52    1033.17  1162.66  12.85   499.11   498.16     0.00
HUB   51.92    50.47     1.45    679.80   863.90   12.85   499.11   498.16     0.00

Station 2. ROTOR EXIT CONDITIONS, STAGE 1   SOLUTION IS CONVERGED
B2 axial  THK      AeroB1    Blades2   Area2     Area2UB
0.7000    0.0500   0.9500    32        2903.6260  3083.4636

```

	R2	C2	Cu2	Cm2	Ao2	Mach2	Chord	AxChord	Rcircle
TIP	37.50	643.01	451.88	457.46	1179.20	0.55	12.22	7.24	200.12
MEAN	30.26	665.63	441.00	498.58	1157.03	0.58	10.91	7.55	61.39
HUB	20.63	679.93	373.46	568.18	1123.82	0.61	10.11	8.04	22.25

	U2	W2	Wu2	MachRel2	DelRCu	Eff2uC	Eff2incC	AvgREff	Ws1/W2
TIP	1235.69	907.54	783.82	0.77	16952.53	0.92	0.91		
MEAN	997.26	747.00	556.26	0.65	13352.30	0.92	0.91	0.91	1.82
HUB	679.80	645.50	306.33	0.57	7708.32	0.92	0.91		

	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
TIP	25.11	1.66	20.51	612.56	1.17	578.06	541.84	0.00		
MEAN	22.68	1.50	18.12	593.50	1.14	556.53	533.72	0.00	589.89	1.48
HUB	19.21	1.27	15.00	563.61	1.08	525.04	519.92	0.00		

	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter
TIP	44.65	59.73	56.50	3.23	0.93	0.45	1.66	0.00010	67
MEAN	41.49	48.13	44.00	4.13	0.93	0.46	1.77	0.00027	57
HUB	33.32	28.33	24.20	4.13	0.93	0.34	2.50	0.00002	27

Station 3. STATOR ROW #1  
blockage3 XBladeGap Vane# Area3 Tt3 TwetBulb3 MeltFract Area3UB  
0.9500 2.0099 33 2895.0393 589.8918 532.1749 0.0000 3080.8125

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE  
R3m C3 Cu3 Cm3 Ao3 Mach3 cp 2-3 Stat Ax Chd  
30.2642 672.7497 441.0006 508.0459 1152.3959 0.5838 -0.0955 9.3931

Pt3 Ps3 Ts3 RH3 FloAlpha3 VaneAlpha3 Incid3  
22.2858 17.6896 552.1353 0.0000 40.9591 40.0000 0.9591

Station 4. STATOR / VANED DIFFUSER  
RCG Cth Cuth Cmth Aoth Machth cp 2-Th Stat Chord  
30.2642 559.8395 154.3127 538.1523 1164.3728 0.4808 0.2468 9.9960

BlockageTh PtTh PsTh TsTh TwetBulbTh AreaTh w2-Th DiffFact4  
0.9500 22.2274 18.9730 563.7550 532.1224 2738.7583 0.0231 0.5175

Station 5. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):  
R4 C4 Cu4 Cm4 Ao4 Mach4 cp 3-4 Stator Gap  
30.2642 482.1255 0.0000 482.1255 1171.2854 0.4116 0.4426 2.3483

Blockage4 Ps4 Ts4 TwetBulb4 VaneAlpha4 Vane Thk4 w2-40D cp 2-4  
0.9500 19.7240 570.5114 532.0667 0.0000 0.0600 0.0369 0.4153

STAGE EXIT CONDITIONS, STAGE 1  
Eff4 Pt4 PR TR Tt4 Del T Ns Ns nondim  
0.9037 22.1658 1.4673 1.1283 589.8918 67.0919 350.0365 2.7135

Del Enthalpy Del\_H/U^2 GHP Reynolds# Specif Humid Area4 Area4UB  
417700.906 0.274 20849.992 5138243.0 0.00000 2827.08154 3080.81274

OVERALL EXIT CONDITIONS; ALL 1 STAGES  
Del Enthal DelHT/U1^2 GHP MassFloSlcor OPR Efficiency RotorInc TR AxHubLen  
417700.91 0.2497 20849.9922 862.893 1.4673 0.90366 8.5213 1.1283 21.791

geometry.out

	R1	Beta1	R2	Beta2	Sol	Rc	Chord	AxC	X1	Y1	X2	Y2	Theta
TIP	39.250	60.000	37.500	56.500	1.660	200.120	12.223	6.432	173.309	100.060	166.878	110.454	3.500
MID	31.354	54.200	30.264	44.000	1.773	61.393	10.915	7.146	49.793	35.912	42.647	44.162	10.200
HUB	20.630	50.470	20.630	24.200	2.496	22.248	10.112	8.040	17.160	14.160	9.120	20.293	26.270

Blade > Stator Gap (2->3) 2.009935

	R3	Beta3	R4	Beta4	Sol	Rc	Chord	AxC	X3	Y3	X4	Y4	Theta
TIP	37.500	40.000	37.500	0.000	1.400	14.613	9.996	9.393	9.393	11.194	0.000	14.613	40.000
MID	30.264	40.000	30.264	0.000	1.682	14.613	9.996	9.393	9.393	11.194	0.000	14.613	40.000
HUB	20.630	40.000	20.630	0.000	2.468	14.613	9.996	9.393	9.393	11.194	0.000	14.613	40.000

Stator > Blade Gap (4->1) 2.348287  
Cumul Axial Length (1->5) Stage 21.79111 1

### 7.1.3 Fan Core and Four-Stage Low-Pressure Compressor (No Ice Crystals)

The input and output files in this section are for the analysis of the fan core and four-stage LPC without ice crystal ingestion for this example case.

#### 7.1.3.1 Input File

comdes.dat

```

Fan-CORE + 4 LPC Stages Bypass Ratio = 5.0 Design Point
176.664 15.107 522.81 3776.0 Wact Pt1 Tt1 RPM
5 5 0 0.0 Fluid Stages IDUMMY DUMMY
5.00 46.900 0.0021 0.0 DropDia InDist RHinlet IWC ***icing***
14.17 0.0 469.58 0.0 Psin DUMMY Tsin DUMMY ***icing***
159.42 0.0 40000. 0 Vin DUMMY Altitude IDUMMY ***icing***
20.63 12.5100 20.63 15.00 1 Rltip Rlhub R2tip R2hub
0.7000 0.700 32 0.050 1.0 Blaxial B2axial NBLAD THK CFS
0.0 0.0000 0.000 1.000 0.9 StatorT StatorM StatorH FSGV CFMid
0.9800 0.9500 1.0000 1.000 0.9 AeroB1 AeroB2 POTS POTH CFHub
0.915 0.915 0.915 0.0000 EfficiT EfficiM EfficiH BLEED
0.000 0.000 0.000 0.0000 DUMMY DUMMY DUMMY DUMMY
0.9250 0.035 1.8 0.00 SlipF DiffLoss Solidity RPM2
50.47 47.2 38.62 2.0 BetaBlade1T BetaBlade1M BetaBlade1H expo3
24.2 12.70 -9.3 0 BetaBlade2T BetaBlade2M BetaBlade2H stat2
20.617 15.102 0.0 35.400 R3s R3h B3X BetB3
33 0.06 0.95 0.01 Blan3 Blath3 Blok3 Wbar23
00.0 0.000 0.95 0.015 RCG Athrt BlokTH Wbar2T
20.573 15.19 0.0 0.0 R4S R4H B4X BetB4
33 0.06 0.95 1.40 Blan4 Blath4 Blok4 SolidStat
20.55 15.21 20.42 15.223 2 Rltip Rlhub R2tip R2hub
0.5000 0.500 56 0.050 1.0 Blaxial B2axial NBLAD THK CFS
0.0 0.0000 0.000 1.000 0.9 StatorT StatorM StatorH FSGV CFMid
0.9800 0.9500 1.0000 1.000 0.9 AeroB1 AeroB2 POTS POTH CFHub
0.915 0.915 0.915 0.0000 EfficiT EfficiM EfficiH BLEED
0.000 0.000 0.000 0.0000 DUMMY DUMMY DUMMY DUMMY
0.9250 0.035 1.4 0.00 SlipF DiffLoss Solidity RPM2
46.36 42.3 37.84 0.00 BetaBlade1T BetaBlade1M BetaBlade1H expo3
31.50 23.50 6.50 0 BetaBlade2T BetaBlade2M BetaBlade2H stat2
20.343 15.207 0.0 30.600 R3s R3h B3X BetB3
73 0.06 0.95 0.01 Blan3 Blath3 Blok3 Wbar23
00.0 0.000 0.95 0.015 RCG Athrt BlokTH Wbar2T
20.170 15.117 0.0 0.0 R4S R4H B4X BetB4
73 0.06 0.95 1.40 Blan4 Blath4 Blok4 SolidStat
20.07 15.05 19.81 14.85 3 Rltip Rlhub R2tip R2hub
0.5000 0.500 56 0.050 1.0 Blaxial B2axial NBLAD THK CFS
0.0 0.0000 0.000 1.000 0.9 StatorT StatorM StatorH FSGV CFMid
0.9800 0.9500 1.0000 1.000 0.9 AeroB1 AeroB2 POTS POTH CFHub
0.915 0.915 0.915 0.0000 EfficiT EfficiM EfficiH BLEED
0.000 0.000 0.000 0.0000 DUMMY DUMMY DUMMY DUMMY
0.9250 0.035 1.4 0.00 SlipF DiffLoss Solidity RPM2
46.36 43.4 38.84 0.00 BetaBlade1T BetaBlade1M BetaBlade1H expo3
31.50 23.50 6.50 0 BetaBlade2T BetaBlade2M BetaBlade2H stat2
19.690 14.741 0.0 31.500 R3s R3h B3X BetB3
73 0.06 0.95 0.01 Blan3 Blath3 Blok3 Wbar23
00.0 0.000 0.95 0.015 RCG Athrt BlokTH Wbar2T
19.411 14.474 0.0 0.0 R4S R4H B4X BetB4
73 0.06 0.95 1.40 Blan4 Blath4 Blok4 SolidStat
19.26 14.32 18.88 13.89 4 Rltip Rlhub R2tip R2hub
0.5000 0.500 56 0.050 1.0 Blaxial B2axial NBLAD THK CFS
0.0 0.0000 0.000 1.000 0.9 StatorT StatorM StatorH FSGV CFMid
0.9800 0.9500 1.0000 1.000 0.9 AeroB1 AeroB2 POTS POTH CFHub
0.915 0.915 0.915 0.0000 EfficiT EfficiM EfficiH BLEED
0.000 0.000 0.000 0.0000 DUMMY DUMMY DUMMY DUMMY
0.9250 0.035 1.4 0.00 SlipF DiffLoss Solidity RPM2
46.36 43.8 37.84 0.00 BetaBlade1T BetaBlade1M BetaBlade1H expo3
31.50 23.50 6.50 0 BetaBlade2T BetaBlade2M BetaBlade2H stat2
18.842 13.720 0.0 32.400 R3s R3h B3X BetB3
73 0.06 0.95 0.01 Blan3 Blath3 Blok3 Wbar23
00.0 0.000 0.95 0.015 RCG Athrt BlokTH Wbar2T
18.485 13.284 0.0 0.0 R4S R4H B4X BetB4
73 0.06 0.95 1.40 Blan4 Blath4 Blok4 SolidStat
18.32 13.07 17.94 12.59 5 Rltip Rlhub R2tip R2hub
0.5000 0.500 56 0.050 1.0 Blaxial B2axial NBLAD THK CFS
0.0 0.0000 0.000 1.000 0.9 StatorT StatorM StatorH FSGV CFMid
0.9800 0.9500 1.0000 1.000 0.9 AeroB1 AeroB2 POTS POTH CFHub
0.915 0.915 0.915 0.0000 EfficiT EfficiM EfficiH BLEED
0.000 0.000 0.000 0.0000 DUMMY DUMMY DUMMY DUMMY
0.9250 0.035 1.4 0.00 SlipF DiffLoss Solidity RPM2
47.36 44.8 38.84 0.00 BetaBlade1T BetaBlade1M BetaBlade1H expo3

```

31.50	23.50	6.50	0	BetaBlade2T	BetaBlade2M	BetaBlade2H	stat2
17.689	12.260	0.0	33.000	R3s	R3h	B3X	BetB3
73	0.06	0.95	0.02	Blan3	Blath3	Blok3	Wbar23
00.0	0.000	0.95	0.03	RCG	Athrt	BlokTH	Wbar2T
17.239	11.622	0.0	0.0	R4S	R4H	B4X	BetB4
73	0.06	0.95	1.40	Blan4	Blath4	Blok4	SolidStat
17.239	11.622	0.00	2.0	R7S	R7h	B7X	Length67
4	0.01	0.98	0.000	Blan7	Blath7	Blok7	Wbar67
17.239	11.622	0.0	2.0	R8S	R8h	B8X	Length78
4	0.01	0.98	0.00	Blan8	Blath8	Blok8	Wbar78

### 7.1.3.2 Output Files

#### comdes.out

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*****
***** AXIAL & CENTRIFUGAL COMPRESSOR BLADE DESIGN 1-D ****
***** COMDES-MELT Release Version 1 *****
***** Gasplus Fluid Properties *****
*****

Fan-CORE + 4 LPC Stages Bypass Ratio = 5.0 Design Point

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 1
BLEED = 0.000 DPInc = 5.912 EfDer = 0.999 SH = 0.210000E-02

W act RPM act Pt Tt POTS POTH AeroBl
176.6640 3776.0000 15.1069 522.8100 1.0000 1.0000 0.9800

W Kg/sec Wdry H2Ovap_Lb/s H2Ovap_g/m^3 m^3/min IWAR IWC
80.3018 176.2930 0.3710 2.4438 4140.3765 0.0000 0.0000

W cor RPM cor GAMMA Cp R Blades THK ArealUB
172.5842 3761.1287 1.3987 0.2502 53.4164 32 0.0500 848.5279

CFM SCFM Al/A* Areal A* AthrRotor ChokeMargin AnnularArea
146216.000 138504.844 1.6521 831.5574 503.3270 619.7536 1.2313 845.3924

Station 1. ROTOR LEADING EDGE CONDITIONS, STAGE 1

R1 Stator Alfa C1 CU1 Cm1 Mabs Mrel Ulcor FlowCoeff
TIP 20.63 0.00 -0.02 422.00 -0.15 422.00 0.38 0.72 677.10 0.621
MEAN 17.06 0.00 -0.02 422.00 -0.15 422.00 0.38 0.64
HUB 12.51 0.00 -0.02 422.00 -0.15 422.00 0.38 0.53

BetaFlo BetaBlade Incid U1 W1 Ps1 Ts1 TwetBulb1 RH
TIP 58.17 50.47 7.70 679.80 800.25 13.66 508.04 506.03 27.56
MEAN 53.11 47.20 5.91 562.16 703.05 13.66 508.04 506.03 27.56
HUB 44.34 38.62 5.72 412.23 590.03 13.66 508.04 506.03 27.56

Station 2. ROTOR EXIT CONDITIONS, STAGE 1 SOLUTION IS CONVERGED
B2 axial THK AeroBl Blades2 Area2 Area2UB
0.7000 0.0500 0.9500 32 594.6702 635.0460

R2 C2 Cu2 Cm2 Ao2 Mach2 Chord AxChord Rcircle
TIP 20.63 669.55 381.55 550.20 1125.59 0.59 7.29 6.52 16.04
MEAN 18.04 704.82 422.65 564.03 1119.89 0.63 7.43 6.80 12.53
HUB 15.00 811.28 554.24 592.44 1109.37 0.73 7.49 7.25 9.22

U2 W2 Wu2 MachRel2 DelRCu Eff2uC Eff2incC AvgREff Ws1/W2
TIP 679.80 625.83 298.25 0.56 7874.33 0.92 0.91
MEAN 594.32 589.58 171.67 0.53 7625.35 0.92 0.91 0.91 1.33
HUB 494.28 595.47 59.96 0.54 8315.36 0.92 0.91

Pt2 PR Ps2 Tt2 TR Ts2 TwetBulb2 RH Tt2avg PR2avg
TIP 19.31 1.28 15.20 564.24 1.08 527.06 529.83 15.47
MEAN 19.16 1.27 14.68 562.93 1.08 521.73 530.39 17.99 564.57 1.28
HUB 19.57 1.30 13.71 566.56 1.08 511.97 536.08 23.90

Alfa2 Beta FLO Beta BLADE Deviat Slip F. DiffFct Solidity Convergence Iter
TIP 34.74 28.46 24.20 4.26 0.93 0.35 1.80 0.00047 22
MEAN 36.85 16.93 12.70 4.23 0.93 0.30 2.22 0.00111 35
HUB 43.09 -5.78 -9.30 3.52 0.93 0.16 3.05 0.00122 42

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Station 3. STATOR ROW #1  
 blockage3 XBladeGap Vane# Area3 Tt3 TwetBulb3 MeltFract Area3UB  
 0.9500 1.8117 33 577.5464 564.5723 531.4438 0.0000 618.8633

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE  
 R3m C3 Cu3 Cm3 Ao3 Mach3 cp 2-3 Stat Ax Chd  
 18.0711 719.5446 421.8260 582.9299 1119.7871 0.6426 -0.0119 5.2355

Pt3 Ps3 Ts3 RH3 FloAlpha3 VaneAlpha3 Incid3  
 19.2975 14.6213 521.6359 17.9838 35.8907 35.4000 0.4907

Station 4. STATOR / VANED DIFFUSER  
 RCG Cth Cuth Cmth Aoth Machth cp 2-Th Stat Chord  
 18.0711 595.8393 145.7605 577.7355 1134.1595 0.5254 0.2908 5.4957

BlockageTh PtTh PsTh TsTh TwetBulbTh AreaTh w2-Th DiffFact4  
 0.9500 19.2240 15.9310 535.1331 528.1395 577.5480 0.0253 0.3999

Station 5. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):  
 R4 C4 Cu4 Cm4 Ao4 Mach4 cp 3-4 Stator Gap  
 18.0829 582.6066 0.0000 582.6066 1135.4991 0.5131 0.2968 1.3089

Blockage4 Ps4 Ts4 TwetBulb4 VaneAlpha4 Vane Thk4 w2-40D cp 2-4  
 0.9500 16.0090 536.4304 527.8182 0.0000 0.0600 0.0395 0.3070

STAGE EXIT CONDITIONS, STAGE 1  
 Eff4 Pt4 PR TR Tt4 Del T Ns Ns nondim  
 0.8763 19.1552 1.2680 1.0799 564.5723 41.7623 217.6874 1.6875

Del Enthalpy Del\_H/U^2 GHP Reynolds# Specif Humid Area4 Area4UB  
 261616.797 0.566 2611.836 1774469.4 0.00210 542.74530 604.79504

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 2  
 BLEED = 0.000 DPInc = 7.014 EfDer = 1.000 SH = 0.210000E-02

W act RPM act Pt Tt POTS POTH AeroBl  
 176.6640 3776.0000 19.1552 564.5723 1.0000 1.0000 0.9800

W Kg/sec Wdry H2Ovap\_Lb/s H2Ovap\_g/m^3 m^3/min IWAR IWC  
 80.3018 176.2930 0.3710 2.7937 3621.7927 0.0000 0.0000

W cor RPM cor GAMMA Cp R Blades THK ArealUB  
 141.4425 3619.3481 1.3985 0.2503 53.4164 56 0.0500 602.5374

CFM SCFM A1/A\* Areal A\* AthrRotor ChokeMargin AnnularArea  
 127902.398 138504.844 1.4524 590.4866 406.5490 474.1340 1.1662 599.9134

Station 6. ROTOR LEADING EDGE CONDITIONS, STAGE 2  
 R1 Stator Alfa C1 CU1 Cm1 Mabs Mrel Ulcor FlowCoeff  
 TIP 20.55 0.00 -0.02 512.30 -0.18 512.30 0.45 0.74 649.05 0.757  
 MEAN 18.08 0.00 -0.02 512.30 -0.18 512.30 0.45 0.69  
 HUB 15.21 0.00 -0.02 512.30 -0.18 512.30 0.45 0.63

BetaFlo BetaBlade Incid U1 W1 Psi Ts1 TwetBulb1 RH  
 TIP 52.90 46.36 6.54 677.16 849.25 16.69 542.82 526.71 10.04  
 MEAN 49.31 42.30 7.01 595.71 785.83 16.69 542.82 526.71 10.04  
 HUB 44.38 37.84 6.54 501.20 716.81 16.69 542.82 526.71 10.04

Station 7. ROTOR EXIT CONDITIONS, STAGE 2 SOLUTION IS CONVERGED  
 B2 axial THK AeroBl Blades2 Area2 Area2UB  
 0.5000 0.0500 0.9500 56 541.5062 584.6252

R2 C2 Cu2 Cm2 Ao2 Mach2 Chord AxChord Rcircle  
 TIP 20.42 599.40 306.91 514.87 1168.17 0.51 3.21 2.81 12.40  
 MEAN 18.01 614.25 321.34 523.50 1164.03 0.53 3.30 2.93 10.11  
 HUB 15.22 668.54 403.25 533.23 1159.96 0.58 3.37 3.12 6.23

U2 W2 Wu2 MachRel2 DelRCu Eff2uC Eff2incC AvgREff Ws1/W2  
 TIP 672.88 631.68 365.97 0.54 6270.63 0.92 0.91  
 MEAN 593.46 590.00 272.12 0.51 5790.48 0.92 0.91 0.91 1.44  
 HUB 501.63 542.23 98.37 0.47 6141.42 0.92 0.91

	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
TIP	22.99	1.20	19.22	597.55	1.06	567.76	541.24	5.38		
MEAN	22.68	1.18	18.77	595.02	1.05	563.74	540.47	5.91	596.48	1.19
HUB	22.91	1.20	18.30	596.87	1.06	559.82	542.02	6.48		

	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter
TIP	30.80	35.41	31.50	3.91	0.93	0.38	1.40	0.00042	22
MEAN	31.54	27.47	23.50	3.97	0.93	0.37	1.63	0.00098	31
HUB	37.10	10.45	6.50	3.95	0.93	0.39	1.97	0.00051	33

Station 8. STATOR ROW #1

blockage3	XBladeGap	Vane#	Area3	Tt3	TwetBulb3	MeltFract	Area3UB
0.9500	0.7798	73	523.5558	596.4807	541.2429	0.0000	573.6071

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE

R3m	C3	Cu3	Cm3	Ao3	Mach3	cp 2-3	Stat Ax Chd
17.9595	632.0330	322.2403	543.7158	1163.5891	0.5432	-0.0221	2.3644

Pt3	Ps3	Ts3	RH3	FloAlpha3	VaneAlpha3	Incid3
22.8212	18.6760	563.3723	5.9483	30.6537	30.6000	0.0537

Station 9. STATOR / VANED DIFFUSER

RCG	Cth	Cuth	Cmth	Aoth	Machth	cp 2-Th	Stat Chord
17.9595	547.7831	116.1339	535.3310	1172.0541	0.4674	0.2078	2.4513

BlockageTh	PtTh	PsTh	TsTh	TwetBulbTh	AreaTh	w2-Th	DiffFact4
0.9500	22.7749	19.6124	571.6121	540.1403	540.8125	0.0213	0.2231

Station 10. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):

R4	C4	Cu4	Cm4	Ao4	Mach4	cp 3-4	Stator Gap
17.8235	606.5960	0.0000	606.5960	1166.2430	0.5201	0.0518	0.5911

Blockage4	Ps4	Ts4	TwetBulb4	VaneAlpha4	Vane Thk4	w2-4OD	cp 2-4
0.9500	18.8909	565.9897	540.7750	0.0000	0.0600	0.0366	0.0319

STAGE EXIT CONDITIONS, STAGE

Eff4	Pt4	PR	TR	Tt4	Del T	Ns	Ns nondim
0.8796	22.7121	1.1857	1.0565	596.4807	31.9084	247.2050	1.9163

Del Enthalpy	Del_H/U^2	GHP	Reynolds#	Specif Humid	Area4	Area4UB
200009.547	0.442	1996.784	1570979.9	0.00210	466.10056	560.16235

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 3

BLEED = 0.000 DPinc = 6.792 EfDer = 1.000 SH = 0.210000E-02

W act	RPM act	Pt	Tt	POTS	POTH	AeroBl
176.6640	3776.0000	22.7121	596.4807	1.0000	1.0000	0.9800

W Kg/sec	Wdry	H2Ovap_Lb/s	H2Ovap_g/m^3	m^3/min	IWAR	IWC
80.3018	176.2930	0.3710	3.1808	3181.0784	0.0000	0.0000

W cor	RPM cor	GAMMA	Cp	R	Blades	THK	ArealUB
117.9940	3659.1458	1.3982	0.2505	53.4164	56	0.0500	556.6108

CFM	SCFM	A1/A*	Areal	A*	AthrRotor	ChokeMargin	AnnularArea
112338.719	138504.844	1.5471	545.4786	352.5860	429.7651	1.2189	553.8704

Station 11. ROTOR LEADING EDGE CONDITIONS, STAGE 3

	R1	Stator	Alfa	C1	CU1	Cm1	Mabs	Mrel	U1cor	FlowCoeff
TIP	20.07	0.00	-0.02	487.28	-0.17	487.28	0.41	0.70	640.86	0.737
MEAN	17.74	0.00	-0.02	487.28	-0.17	487.28	0.41	0.65		
HUB	15.05	0.00	-0.02	487.28	-0.17	487.28	0.41	0.59		

	BetaFlo	BetaBlade	Incid	U1	W1	Ps1	Ts1	TwetBulb1	RH
TIP	53.62	46.36	7.26	661.34	821.61	20.19	576.81	539.52	4.36
MEAN	50.19	43.40	6.79	584.52	761.11	20.19	576.81	539.52	4.36
HUB	45.51	38.84	6.67	495.92	695.38	20.19	576.81	539.52	4.36

Station 12. ROTOR EXIT CONDITIONS, STAGE 3 SOLUTION IS CONVERGED

B2 axial	THK	AeroBl	Blades2	Area2	Area2UB
0.5000	0.0500	0.9500	56	502.4182	542.8196

	R2	C2	Cu2	Cm2	Ao2	Mach2	Chord	AxChord	Rcircle	
TIP	19.81	577.94	301.78	492.89	1200.99	0.48	3.11	2.72	12.03	
MEAN	17.51	591.58	316.23	499.97	1197.34	0.49	3.22	2.84	9.33	
HUB	14.85	643.34	394.77	507.99	1193.74	0.54	3.28	3.03	5.89	
	U2	W2	Wu2	MachRel2	DelRCu	Eff2uC	Eff2incC	AvgREff	Wsl/W2	
TIP	652.78	605.10	351.00	0.50	5981.54	0.92	0.91			
MEAN	576.87	563.83	260.64	0.47	5539.04	0.92	0.91	0.91	1.46	
HUB	489.33	516.71	94.57	0.43	5864.80	0.92	0.91			
	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
TIP	26.80	1.18	22.87	627.92	1.05	600.24	552.24	2.63		
MEAN	26.48	1.17	22.41	625.59	1.05	596.60	551.51	2.84	626.94	1.17
HUB	26.71	1.18	21.93	627.31	1.05	593.01	552.64	3.05		
	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter	
TIP	31.48	35.46	31.50	3.96	0.93	0.39	1.40	0.00035	22	
MEAN	32.31	27.53	23.50	4.03	0.93	0.39	1.62	0.00118	32	
HUB	37.85	10.55	6.50	4.05	0.93	0.40	1.94	0.00104	31	

Station 13. STATOR ROW #1  
 blockage3 XBladeGap Vane# Area3 Tt3 TwetBulb3 MeltFract Area3UB  
 0.9500 0.7565 73 487.9654 626.9401 552.0973 0.0000 535.3244

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE  
 R3m C3 Cu3 Cm3 Ao3 Mach3 cp 2-3 Stat Ax Chd  
 17.3924 606.4476 318.3029 516.1995 1197.1470 0.5066 -0.0156 2.2836  
 Pt3 Ps3 Ts3 RH3 FloAlpha3 VaneAlpha3 Incid3  
 26.6189 22.3465 596.4758 2.8369 31.6591 31.5000 0.1591

Station 14. STATOR / VANED DIFFUSER  
 RCG Cth Cuth Cmth Aoth Machth cp 2-Th Stat Chord  
 17.3924 529.7297 115.5570 516.9721 1204.3553 0.4398 0.2044 2.3726

BlockageTh PtTh PsTh TsTh TwetBulbTh AreaTh w2-Th DiffFact4  
 0.9500 26.5769 23.2743 603.6976 551.4423 498.3970 0.0199 0.2419

Station 15. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):  
 R4 C4 Cu4 Cm4 Ao4 Mach4 cp 3-4 Stator Gap  
 17.1214 574.2238 0.0000 574.2238 1200.2572 0.4784 0.0750 0.5709

Blockage4 Ps4 Ts4 TwetBulb4 VaneAlpha4 Vane Thk4 w2-4OD cp 2-4  
 0.9500 22.6671 599.6335 551.7104 0.0000 0.0600 0.0366 0.0617

STAGE EXIT CONDITIONS, STAGE 3  
 Eff4 Pt4 PR TR Tt4 Del T Ns Ns nondim  
 0.8803 26.5056 1.1670 1.0511 626.9401 30.4594 239.8310 1.8592

Del Enthalpy Del\_H/U^2 GHP Reynolds# Specif Humid Area4 Area4UB  
 191044.703 0.448 1907.284 1730067.9 0.00210 434.74258 525.55774

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 4  
 BLEED = 0.000 DPInc = 6.732 EfDer = 1.000 SH = 0.210000E-02

W act RPM act Pt Tt POTS POTH AeroBl  
 176.6640 3776.0000 26.5056 626.9401 1.0000 1.0000 0.9800

W Kg/sec Wdry H2Ovap\_Lb/s H2Ovap\_g/m^3 m^3/min IWAR IWC  
 80.3018 176.2930 0.3710 3.5772 2828.5518 0.0000 0.0000

W cor RPM cor GAMMA Cp R Blades THK Area1UB  
 100.8453 3668.6252 1.3979 0.2506 53.4164 56 0.0500 523.8064

CFM SCFM A1/A\* Areal A\* AthrRotor ChokeMargin AnnularArea  
 99889.359 138504.844 1.6566 513.3303 309.8729 401.0588 1.2943 521.1437

Station 16. ROTOR LEADING EDGE CONDITIONS, STAGE 4

	R1	Stator	Alfa	Cl	CU1	Cm1	Mabs	Mrel	U1cor	FlowCoeff
TIP	19.26	0.00	-0.02	460.59	-0.16	460.59	0.38	0.65	616.59	0.726
MEAN	16.97	0.00	-0.02	460.59	-0.16	460.59	0.38	0.60		
HUB	14.32	0.00	-0.02	460.59	-0.16	460.59	0.38	0.55		



	BetaFlo	BetaBlade	Incid	U1	W1	Ps1	Ts1	TwetBulb1	RH
TIP	54.04	46.36	7.68	634.65	784.30	23.99	609.37	550.96	2.19
MEAN	50.53	43.80	6.73	559.22	724.60	23.99	609.37	550.96	2.19
HUB	45.70	37.84	7.86	471.87	659.51	23.99	609.37	550.96	2.19

Station 17. ROTOR EXIT CONDITIONS, STAGE 4 SOLUTION IS CONVERGED

B2 axial	THK	AeroBl	Blades2	Area2	Area2UB					
0.5000	0.0500	0.9500	56	477.1383	516.2928					
TIP	R2	C2	Cu2	Cm2	Ao2	Mach2	Chord	AxChord	Rcircle	
18.88	18.88	549.35	289.30	467.00	1230.88	0.45	2.97	2.60	11.47	
MEAN	16.57	560.00	299.43	473.23	1227.36	0.46	3.08	2.71	8.74	
HUB	13.89	605.41	368.68	480.21	1223.88	0.49	3.11	2.88	5.76	
TIP	U2	W2	Wu2	MachRel2	DelRCu	Eff2uC	Eff2incC	AvgREff	Wsl/W2	
622.13	622.13	573.47	332.83	0.47	5464.98	0.92	0.91			
MEAN	546.14	533.68	246.71	0.43	4965.36	0.92	0.91	0.91	1.47	
HUB	457.70	488.39	89.03	0.40	5123.17	0.92	0.91			
TIP	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
30.62	30.62	1.16	26.71	655.64	1.05	630.66	561.80	1.47		
MEAN	30.22	1.14	26.21	653.02	1.04	627.05	560.95	1.57	654.17	1.15
HUB	30.35	1.14	25.68	653.85	1.04	623.50	561.52	1.67		
TIP	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter	
31.78	31.78	35.48	31.50	3.98	0.93	0.40	1.40	0.00036	21	
MEAN	32.32	27.53	23.50	4.03	0.93	0.39	1.62	0.00135	29	
HUB	37.51	10.50	6.50	4.00	0.93	0.40	1.94	0.00176	30	

Station 18. STATOR ROW #1

blockage3	XBladeGap	Vane#	Area3	Tt3	TwetBulb3	MeltFract	Area3UB
0.9500	0.7209	73	476.4520	654.1712	561.3143	0.0000	523.9628

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE

R3m	C3	Cu3	Cm3	Ao3	Mach3	cp 2-3	Stat Ax Chd
16.4812	560.4597	301.1112	472.7019	1228.3756	0.4563	0.0271	2.1803
Pt3	Ps3	Ts3	RH3	FloAlpha3	VaneAlpha3	Incid3	
30.3530	26.3204	628.1697	1.5355	32.4971	32.4000	0.0971	

Station 19. STATOR / VANED DIFFUSER

RCG	Cth	Cuth	Cmth	Aoth	Machth	cp 2-Th	Stat Chord
16.4812	495.6020	111.1490	482.9775	1233.8801	0.4017	0.2233	2.2705

BlockageTh	PtTh	PsTh	TsTh	TwetBulbTh	AreaTh	w2-Th	DiffFact4
0.9500	30.3199	27.1353	633.8417	560.9500	480.4178	0.0179	0.2570

Station 20. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):

R4	C4	Cu4	Cm4	Ao4	Mach4	cp 3-4	Stator Gap
16.0960	524.9850	0.0000	524.9850	1231.4290	0.4263	0.0927	0.5451

Blockage4	Ps4	Ts4	TwetBulb4	VaneAlpha4	Vane Thk4	w2-40D	cp 2-4
0.9500	26.6943	631.3631	561.0234	0.0000	0.0600	0.0366	0.1182

STAGE EXIT CONDITIONS, STAGE 4

Eff4	Pt4	PR	TR	Tt4	Del T	Ns	Ns nondim
0.8799	30.2414	1.1409	1.0434	654.1712	27.2311	245.8912	1.9061

Del Enthalpy	Del_H/U^2	GHP	Reynolds#	Specif Humid	Area4	Area4UB
170918.797	0.442	1706.358	1894891.5	0.00210	425.14447	519.08722

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 5

BLEED = 0.000 DPInc = 6.641 EfDer = 1.000 SH = 0.210000E-02

W act	RPM act	Pt	Tt	POTS	POTH	AeroBl
176.6640	3776.0000	30.2414	654.1712	1.0000	1.0000	0.9800

W Kg/sec	Wdry	H2Ovap_Lb/s	H2Ovap_g/m^3	m^3/min	IWAR	IWC
80.3018	176.2930	0.3710	3.9713	2547.8423	0.0000	0.0000

W cor	RPM cor	GAMMA	Cp	R	Blades	THK	Area1UB
88.0665	3682.0144	1.3974	0.2508	53.4164	56	0.0500	520.0693

CFM	SCFM	A1/A*	Areal	A*	AthrRotor	ChokeMargin	AnnularArea
89976.195	138504.844	1.8360	509.6679	277.5936	390.2764	1.4059	517.7266

Station 21. ROTOR LEADING EDGE CONDITIONS, STAGE 5

	R1	Stator	Alfa	C1	CU1	Cm1	Mabs	Mrel	U1cor	FlowCoeff
TIP	18.32	0.00	-0.02	418.10	-0.14	418.10	0.34	0.59	588.63	0.693
MEAN	15.91	0.00	-0.02	418.10	-0.14	418.10	0.34	0.54		
HUB	13.07	0.00	-0.02	418.10	-0.14	418.10	0.34	0.48		
	BetaFlo	BetaBlade	Incid	U1	W1	Ps1	Ts1	TwetBulb1	RH	
TIP	55.30	47.36	7.94	603.68	734.44	27.96	639.71	560.57	1.26	
MEAN	51.44	44.80	6.64	524.36	670.75	27.96	639.70	560.57	1.26	
HUB	45.86	38.84	7.02	430.68	600.34	27.96	639.71	560.57	1.26	

Station 22. ROTOR EXIT CONDITIONS, STAGE 5 SOLUTION IS CONVERGED

B2 axial	THK	AeroBl	Blades2	Area2	Area2UB					
0.5000	0.0500	0.9500	56	475.3082	515.3698					
	R2	C2	Cu2	Cm2	Ao2	Mach2	Chord	AxChord	Rcircle	
TIP	17.94	513.03	285.69	426.13	1258.35	0.41	2.82	2.45	10.21	
MEAN	15.50	517.44	284.44	432.25	1254.39	0.41	2.92	2.55	7.91	
HUB	12.59	551.39	333.76	438.90	1250.47	0.44	2.95	2.72	5.29	
	U2	W2	Wu2	MachRel2	DelRCu	Eff2uC	Eff2incC	AvgREff	Wsl/W2	
TIP	591.16	524.31	305.47	0.42	5127.88	0.92	0.91			
MEAN	510.67	487.87	226.23	0.39	4410.43	0.92	0.91	0.91	1.51	
HUB	414.86	446.33	81.10	0.36	4203.98	0.92	0.91			
	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
TIP	34.44	1.14	30.72	681.08	1.04	659.31	570.22	0.91		
MEAN	33.83	1.12	30.09	677.32	1.04	655.16	568.97	0.97	678.21	1.12
HUB	33.65	1.11	29.45	676.23	1.03	651.08	568.75	1.03		
	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter	
TIP	33.84	35.63	31.50	4.13	0.93	0.42	1.40	0.00021	21	
MEAN	33.35	27.63	23.50	4.13	0.93	0.40	1.64	0.00136	31	
HUB	37.25	10.47	6.50	3.97	0.93	0.39	2.01	0.00284	29	

Station 23. STATOR ROW #1

blockage3	XBladeGap	Vane#	Area3	Tt3	TwetBulb3	MeltFract	Area3UB
0.9500	0.6802	73	462.6711	678.2124	569.2645	0.0000	510.8013

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE

R3m	C3	Cu3	Cm3	Ao3	Mach3	cp 2-3	Stat Ax Chd
15.2185	531.6449	289.6560	445.8090	1253.9929	0.4240	-0.0345	2.0437
Pt3	Ps3	Ts3	RH3	FloAlpha3	VaneAlpha3	Incid3	
33.8936	29.9593	654.8327	0.9710	33.0130	33.0000	0.0130	

Station 24. STATOR / VANED DIFFUSER

RCG	Cth	Cuth	Cmth	Aoth	Machth	cp 2-Th	Stat Chord
15.2185	475.6450	108.6139	463.0779	1258.4282	0.3780	0.1500	2.1315
BlockageTh	PtTh	PsTh	TsTh	TwetBulbTh	AreaTh	w2-Th	DiffFact4
0.9500	33.8421	30.6703	659.5006	569.0118	461.2568	0.0332	0.2603

Station 25. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):

R4	C4	Cu4	Cm4	Ao4	Mach4	cp 3-4	Stator Gap
14.7013	498.0562	0.0000	498.0562	1256.6731	0.3963	0.1030	0.5109
Blockage4	Ps4	Ts4	TwetBulb4	VaneAlpha4	Vane Thk4	w2-40D	cp 2-4
0.9500	30.3644	657.6986	569.0802	0.0000	0.0600	0.0366	0.0712

STAGE EXIT CONDITIONS, STAGE 5

Eff4	Pt4	PR	TR	Tt4	Del T	Ns	Ns nondim
0.8811	33.8291	1.1186	1.0368	678.2124	24.0412	256.1537	1.9857
Del Enthalpy	Del_H/U^2	GHP	Reynolds#	Specif Humid	Area4	Area4UB	
151012.625	0.432	1507.626	2112125.2	0.00210	410.39981	509.29071	

OVERALL EXIT CONDITIONS; ALL 5 STAGES

Del Enthal	DelHT/U1^2	GHP	MassFloSicor	OPR	Efficiency	RotorInc	TR	AxHubLen
974602.50	2.1090	9729.8877	172.584	2.2393	0.86776	5.9125	1.2972	41.379

geometry.out

	R1	Beta1	R2	Beta2	Sol	Rc	Chord	AxC	X1	Y1	X2	Y2	Theta
TIP	20.630	50.470	20.630	24.200	1.800	16.043	7.291	5.797	12.374	10.211	6.576	14.633	26.270
MID	17.060	47.200	18.036	12.700	2.219	12.535	7.434	6.441	9.197	8.517	2.756	12.228	34.500
HUB	12.510	38.620	15.000	-9.300	3.049	9.222	7.490	7.247	5.756	7.206	1.490	9.101	47.920

Blade > Stator Gap (2->3) 1.811653

	R3	Beta3	R4	Beta4	Sol	Rc	Chord	AxC	X3	Y3	X4	Y4	Theta
TIP	20.617	35.400	20.573	0.000	1.400	9.038	5.496	5.235	5.235	7.367	0.000	9.038	35.400
MID	18.071	35.400	18.083	0.000	1.549	9.038	5.496	5.235	5.235	7.367	0.000	9.038	35.400
HUB	15.102	35.400	15.190	0.000	1.853	9.038	5.496	5.235	5.235	7.367	0.000	9.038	35.400

Stator > Blade Gap (4->1) 1.308875

Cumul Axial Length (1->5) Stage 15.60264 1

	R1	Beta1	R2	Beta2	Sol	Rc	Chord	AxC	X1	Y1	X2	Y2	Theta
TIP	20.550	46.360	20.420	31.500	1.400	12.402	3.208	2.495	8.975	8.559	6.480	10.575	14.860
MID	18.078	42.300	18.010	23.500	1.628	10.109	3.302	2.772	6.803	7.477	4.031	9.270	18.800
HUB	15.210	37.840	15.223	6.500	1.974	6.235	3.368	3.119	3.825	4.924	0.706	6.195	31.340

Blade > Stator Gap (2->3) 0.7797533

	R3	Beta3	R4	Beta4	Sol	Rc	Chord	AxC	X3	Y3	X4	Y4	Theta
TIP	20.343	30.600	20.170	0.000	1.400	4.645	2.451	2.364	2.364	3.998	0.000	4.645	30.600
MID	17.960	30.600	17.823	0.000	1.217	4.645	2.451	2.364	2.364	3.998	0.000	4.645	30.600
HUB	15.207	30.600	15.117	0.000	1.437	4.645	2.451	2.364	2.364	3.998	0.000	4.645	30.600

Stator > Blade Gap (4->1) 0.5911098

Cumul Axial Length (1->5) Stage 22.45695 2

	R1	Beta1	R2	Beta2	Sol	Rc	Chord	AxC	X1	Y1	X2	Y2	Theta
TIP	20.070	46.360	19.810	31.500	1.400	12.032	3.112	2.421	8.707	8.303	6.287	10.259	14.860
MID	17.738	43.400	17.507	23.500	1.620	9.328	3.224	2.690	6.409	6.778	3.720	8.554	19.900
HUB	15.050	38.840	14.850	6.500	1.942	5.887	3.279	3.026	3.692	4.586	0.666	5.850	32.340

Blade > Stator Gap (2->3) 0.7564600

	R3	Beta3	R4	Beta4	Sol	Rc	Chord	AxC	X3	Y3	X4	Y4	Theta
TIP	19.690	31.500	19.411	0.000	1.400	4.370	2.373	2.284	2.284	3.726	0.000	4.370	31.500
MID	17.392	31.500	17.121	0.000	1.216	4.370	2.373	2.284	2.284	3.726	0.000	4.370	31.500
HUB	14.741	31.500	14.474	0.000	1.435	4.370	2.373	2.284	2.284	3.726	0.000	4.370	31.500

Stator > Blade Gap (4->1) 0.5708885

Cumul Axial Length (1->5) Stage 29.09369 3

	R1	Beta1	R2	Beta2	Sol	Rc	Chord	AxC	X1	Y1	X2	Y2	Theta
TIP	19.260	46.360	18.880	31.500	1.400	11.467	2.966	2.307	8.298	7.914	5.991	9.777	14.860
MID	16.971	43.800	16.574	23.500	1.617	8.737	3.079	2.563	6.047	6.306	3.484	8.012	20.300
HUB	14.320	37.840	13.890	6.500	1.938	5.765	3.114	2.884	3.536	4.552	0.653	5.728	31.340

Blade > Stator Gap (2->3) 0.7209471

	R3	Beta3	R4	Beta4	Sol	Rc	Chord	AxC	X3	Y3	X4	Y4	Theta
TIP	18.842	32.400	18.485	0.000	1.400	4.069	2.270	2.180	2.180	3.436	0.000	4.069	32.400
MID	16.481	32.400	16.096	0.000	1.228	4.069	2.270	2.180	2.180	3.436	0.000	4.069	32.400
HUB	13.720	32.400	13.284	0.000	1.475	4.069	2.270	2.180	2.180	3.436	0.000	4.069	32.400

Stator > Blade Gap (4->1) 0.5450748

Cumul Axial Length (1->5) Stage 35.42380 4

	R1	Beta1	R2	Beta2	Sol	Rc	Chord	AxC	X1	Y1	X2	Y2	Theta
TIP	18.320	47.360	17.940	31.500	1.400	10.213	2.818	2.177	7.513	6.918	5.336	8.708	15.860
MID	15.913	44.800	15.498	23.500	1.637	7.906	2.922	2.418	5.571	5.610	3.153	7.251	21.300
HUB	13.070	38.840	12.590	6.500	2.011	5.294	2.949	2.721	3.320	4.123	0.599	5.260	32.340

Blade > Stator Gap (2->3) 0.6801976

	R3	Beta3	R4	Beta4	Sol	Rc	Chord	AxC	X3	Y3	X4	Y4	Theta
TIP	17.689	33.000	17.239	0.000	1.400	3.752	2.132	2.044	2.044	3.147	0.000	3.752	33.000
MID	15.219	33.000	14.701	0.000	1.248	3.752	2.132	2.044	2.044	3.147	0.000	3.752	33.000
HUB	12.260	33.000	11.622	0.000	1.550	3.752	2.132	2.044	2.044	3.147	0.000	3.752	33.000

Stator > Blade Gap (4->1) 0.5109345

Cumul Axial Length (1->5) Stage 41.37946 5

### 7.1.4 Fan Core and Four-Stage Low-Pressure Compressor (With Ice Crystal Ingestion)

The input and output files in this section are those used for the analysis of the fan core and four-stage LPC with ice crystal ingestion for this example case.

With ice crystals: IWC = 2.0 g/m<sup>3</sup>, 5 μm; 40,000 ft altitude, Mach 0.70

Atmospheric conditions: from standard atmosphere (ISA + 25 R):

Pamb (Pstatic)= 2.72 psia

Tamb (Tstatic)= 415 R

Get engine inlet conditions: from compressibility tables (Mach 0.70):

Pt inlet = 3.773 psia

Tt inlet = 455.7 R

Fan N1corrected: 102.7 percent of design

(Full-fan physical tip speed = 1,328 ft/s, below the typical tip speed limit of approximately 1,500 ft/s)

Wcorrected: 100.0 percent of design

#### 7.1.4.1 Input File

comdes.dat

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Fan-CORE + 4 LPC Stages Bypass Ratio = 5.0 Design Wcorrected & N1 corrected
47.26 3.773 455.7 3620.0 Wact Pt1 Tt1 RPM
5 5 0 0.0 Fluid Stages IDUMMY DUMMY
5.00 46.900 100.00 2.0 DropDia InDist RHinlet IWC ***icing***
14.17 0.0 469.58 0.0 Psin DUMMY Tsin DUMMY ***icing***
159.42 0.0 40000. 0 Vin DUMMY Altitude IDUMMY ***icing***
20.63 12.5100 20.63 15.00 1 Rltip Rlhub R2tip R2hub
0.7000 0.700 32 0.050 1.0 Blaxial B2axial NBLAD THK CFS
0.0 0.0000 0.000 1.000 0.9 StatorT StatorM StatorH FSGV CFMid
0.9800 0.9500 1.0000 1.000 0.9 AeroB1 AeroB2 POTS POTH CFHub
0.915 0.915 0.915 0.0000 EfficiT EfficiM EfficiH BLEED
0.000 0.000 0.000 0.0000 DUMMY DUMMY DUMMY DUMMY
0.9250 0.035 1.8 0.00 SlipF DiffLoss Solidity RPM2
50.47 47.2 38.62 2.0 BetaBlade1T BetaBlade1M BetaBlade1H expo3
24.2 12.70 -9.3 0 BetaBlade2T BetaBlade2M BetaBlade2H stat2
20.617 15.102 0.0 35.400 R3s R3h B3X BetB3
33 0.06 0.95 0.01 Blan3 Blath3 Blok3 Wbar23
00.0 0.000 0.95 0.015 RCG Athrt BlokTH Wbar2T
20.573 15.19 0.0 0.0 R4S R4H B4X BetB4
33 0.06 0.95 1.40 Blan4 Blath4 Blok4 SolidStat
20.55 15.21 20.42 15.223 2 Rltip Rlhub R2tip R2hub
0.5000 0.500 56 0.050 1.0 Blaxial B2axial NBLAD THK CFS
0.0 0.0000 0.000 1.000 0.9 StatorT StatorM StatorH FSGV CFMid
0.9800 0.9500 1.0000 1.000 0.9 AeroB1 AeroB2 POTS POTH CFHub
0.915 0.915 0.915 0.0000 EfficiT EfficiM EfficiH BLEED
0.000 0.000 0.000 0.0000 DUMMY DUMMY DUMMY DUMMY
0.9250 0.035 1.4 0.00 SlipF DiffLoss Solidity RPM2
46.36 42.3 37.84 0.00 BetaBlade1T BetaBlade1M BetaBlade1H expo3
31.50 23.50 6.50 0 BetaBlade2T BetaBlade2M BetaBlade2H stat2
20.343 15.207 0.0 30.600 R3s R3h B3X BetB3
73 0.06 0.95 0.01 Blan3 Blath3 Blok3 Wbar23
00.0 0.000 0.95 0.015 RCG Athrt BlokTH Wbar2T
20.170 15.117 0.0 0.0 R4S R4H B4X BetB4
73 0.06 0.95 1.40 Blan4 Blath4 Blok4 SolidStat
20.07 15.05 19.81 14.85 3 Rltip Rlhub R2tip R2hub
0.5000 0.500 56 0.050 1.0 Blaxial B2axial NBLAD THK CFS
0.0 0.0000 0.000 1.000 0.9 StatorT StatorM StatorH FSGV CFMid
0.9800 0.9500 1.0000 1.000 0.9 AeroB1 AeroB2 POTS POTH CFHub
0.915 0.915 0.915 0.0000 EfficiT EfficiM EfficiH BLEED
0.000 0.000 0.000 0.0000 DUMMY DUMMY DUMMY DUMMY
0.9250 0.035 1.4 0.00 SlipF DiffLoss Solidity RPM2
46.36 43.4 38.84 0.00 BetaBlade1T BetaBlade1M BetaBlade1H expo3
31.50 23.50 6.50 0 BetaBlade2T BetaBlade2M BetaBlade2H stat2

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19.690	14.741	0.0	31.500	R3s	R3h	B3X	BetB3		
73	0.06	0.95	0.01	Blan3	Blath3	Blok3	Wbar23		
00.0	0.000	0.95	0.015	RCG	Athrt	BlokTH	Wbar2T		
19.411	14.474	0.0	0.0	R4S	R4H	B4X	BetB4		
73	0.06	0.95	1.40	Blan4	Blath4	Blok4	SolidStat		
19.26	14.32	18.88	13.89	4 Rltip	Rlhub	R2tip	R2hub		
0.5000	0.500	56	0.050	1.0	Blaxial	B2axial	NBLAD	THK	CFS
0.0	0.0000	0.000	1.000	0.9	StatorT	StatorM	StatorH	FSGV	CFMid
0.9800	0.9500	1.0000	1.000	0.9	AeroB1	AeroB2	POTS	POTH	CFHub
0.915	0.915	0.915	0.0000	EfficiT	EfficiM	EfficiH	BLEED		
0.000	0.000	0.000	0.0000	DUMMY	DUMMY	DUMMY	DUMMY		
0.9250	0.035	1.4	0.00	SlipF	DiffLoss	Solidity	RPM2		
46.36	43.8	37.84	0.00	BetaBlade1T	BetaBlade1M	BetaBlade1H	expo3		
31.50	23.50	6.50	0	BetaBlade2T	BetaBlade2M	BetaBlade2H	stat2		
18.842	13.720	0.0	32.400	R3s	R3h	B3X	BetB3		
73	0.06	0.95	0.01	Blan3	Blath3	Blok3	Wbar23		
00.0	0.000	0.95	0.015	RCG	Athrt	BlokTH	Wbar2T		
18.485	13.284	0.0	0.0	R4S	R4H	B4X	BetB4		
73	0.06	0.95	1.40	Blan4	Blath4	Blok4	SolidStat		
18.32	13.07	17.94	12.59	5 Rltip	Rlhub	R2tip	R2hub		
0.5000	0.500	56	0.050	1.0	Blaxial	B2axial	NBLAD	THK	CFS
0.0	0.0000	0.000	1.000	0.9	StatorT	StatorM	StatorH	FSGV	CFMid
0.9800	0.9500	1.0000	1.000	0.9	AeroB1	AeroB2	POTS	POTH	CFHub
0.915	0.915	0.915	0.0000	EfficiT	EfficiM	EfficiH	BLEED		
0.000	0.000	0.000	0.0000	DUMMY	DUMMY	DUMMY	DUMMY		
0.9250	0.035	1.4	0.00	SlipF	DiffLoss	Solidity	RPM2		
47.36	44.8	38.84	0.00	BetaBlade1T	BetaBlade1M	BetaBlade1H	expo3		
31.50	23.50	6.50	0	BetaBlade2T	BetaBlade2M	BetaBlade2H	stat2		
17.689	12.260	0.0	33.000	R3s	R3h	B3X	BetB3		
73	0.06	0.95	0.02	Blan3	Blath3	Blok3	Wbar23		
00.0	0.000	0.95	0.03	RCG	Athrt	BlokTH	Wbar2T		
17.239	11.622	0.0	0.0	R4S	R4H	B4X	BetB4		
73	0.06	0.95	1.40	Blan4	Blath4	Blok4	SolidStat		
17.239	11.622	0.00	2.0	R7S	R7h	B7X	Length67		
4	0.01	0.98	0.000	Blan7	Blath7	Blok7	Wbar67		
17.239	11.622	0.0	2.0	R8S	R8h	B8X	Length78		
4	0.01	0.98	0.00	Blan8	Blath8	Blok8	Wbar78		

## 7.1.4.2 Output Files

### comdes.out

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*****
***** AXIAL & CENTRIFUGAL COMPRESSOR BLADE DESIGN 1-D *****
***** COMDES-MELT Release Version 1 *****
***** Gasplus Fluid Properties *****
*****

Fan-CORE + 4 LPC Stages Bypass Ratio = 5.0 Design Wcorrected & N1 corrected

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 1
BLEED = 0.000 DPinc = 6.654 EfDer = 1.000 SH = 0.139470E-02

W act      RPM act      Pt      Tt      POTS      POTH      AeroBl
47.2600    3620.0000    3.7730  455.7065  1.0000    1.0000    0.9800

W Kg/sec   Wdry      H2Ovap_Lb/s  H2Ovap_g/m^3  m^3/min   IWAR      IWC
21.4818    47.1941    0.0659    0.4653    3863.3096  0.0060    2.0000

W cor      RPM cor      GAMMA     Cp      R      Blades    THK      ArealUB
172.5860   3862.1326   1.4010    0.2490   53.3935   32        0.0500   848.5279

CFM        SCFM        Al/A*     Areal    A*      AthrRotor  ChokeMargin  AnnularArea
136431.484 37062.211  1.6534    831.5574 502.9416 619.7536  1.2323    845.3924

Station 1. ROTOR LEADING EDGE CONDITIONS, STAGE 1

TIP      R1      Stator  Alfa    C1      CU1      Cml      Mabs    Mrel    U1cor   FlowCoeff
20.63    0.00    -0.02   393.76 -0.14   393.76  0.38    0.74    695.28  0.604
MEAN     17.06    0.00    -0.02   393.76 -0.14   393.76  0.38    0.65
HUB      12.51    0.00    -0.02   393.76 -0.14   393.76  0.38    0.54

BetaFlo   BetaBlade  Incid    U1      W1      Ps1     Ts1     TwetBulb1  RH
TIP      58.87    50.47    8.40    651.71  761.55  3.41    442.79    453.81    79.63
MEAN     53.85    47.20    6.65    538.94  667.57  3.41    442.79    453.81    79.63
HUB      45.11    38.62    6.49    395.20  557.97  3.41    442.79    453.81    79.63

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Station 2. ROTOR EXIT CONDITIONS, STAGE 1 SOLUTION IS CONVERGED										
B2 axial	THK	AeroBl	Blades2	Area2	Area2UB					
0.7000	0.0500	0.9500	32	594.6702	635.0460					
	R2	C2	Cu2	Cm2	Ao2	Mach2	Chord	AxChord	Rcircle	
TIP	20.63	631.13	374.55	507.97	1054.13	0.60	7.29	6.52	16.04	
MEAN	18.04	663.46	409.39	522.10	1048.14	0.63	7.43	6.80	12.53	
HUB	15.00	762.28	528.30	549.51	1037.76	0.73	7.49	7.25	9.22	
	U2	W2	Wu2	MachRel2	DelRCu	Eff2uC	Eff2incC	AvgREff	Ws1/W2	
TIP	651.71	578.67	277.16	0.55	7729.72	0.92	0.91			
MEAN	569.77	546.17	160.38	0.52	7386.02	0.92	0.91	0.91	1.36	
HUB	473.86	552.20	54.45	0.53	7926.27	0.92	0.91			
	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
TIP	4.92	1.30	3.86	494.88	1.09	461.69	481.96	40.57		
MEAN	4.86	1.29	3.71	493.13	1.08	456.46	482.79	49.92	494.63	1.30
HUB	4.95	1.31	3.46	495.87	1.09	447.47	490.20	72.12		
	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter	
TIP	36.40	28.62	24.20	4.42	0.93	0.38	1.80	0.00062	25	
MEAN	38.10	17.08	12.70	4.38	0.93	0.32	2.22	0.00127	34	
HUB	43.87	-5.66	-9.30	3.64	0.93	0.18	3.05	0.00010	41	
Station 3. STATOR ROW #1										
Blockage3	XBladeGap	Vane#	Area3	Tt3	TwetBulb3	MeltFract	Area3UB			
0.9500	1.8117	33	577.5464	494.6273	484.0565	0.0000	618.8633			
ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE										
R3m	C3	Cu3	Cm3	Ao3	Mach3	cp 2-3	Stat Ax	Chd		
18.0711	676.6714	408.5916	539.3858	1048.1318	0.6456	-0.0095	5.2355			
Pt3	Ps3	Ts3	RH3	FloAlpha3	VaneAlpha3	Incid3				
4.8949	3.6982	456.5078	50.0089	37.1444	35.4000	1.7444				
Station 4. STATOR / VANED DIFFUSER										
RCG	Cth	Cuth	Cmth	Aoth	Machth	cp 2-Th	Stat	Chord		
18.0711	547.7717	134.0017	531.1284	1063.0168	0.5153	0.3189	5.4957			
BlockageTh	PtTh	PsTh	TsTh	TwetBulbTh	AreaTh	w2-Th	DiffFact4			
0.9500	4.8752	4.0673	469.6565	479.2444	577.5480	0.0260	0.4243			
Station 5. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):										
R4	C4	Cu4	Cm4	Ao4	Mach4	cp 3-4	Stator	Gap		
18.0829	535.6375	0.0000	535.6375	1064.1667	0.5033	0.3247	1.3089			
Blockage4	Ps4	Ts4	TwetBulb4	VaneAlpha4	Vane Thk4	w2-40D	cp 2-4			
0.9500	4.0868	470.7619	479.9468	0.0000	0.0600	0.0394	0.3347			
STAGE EXIT CONDITIONS, STAGE 1										
Eff4	Pt4	PR	TR	Tt4	Del T	Ns	Ns nondim			
0.8772	4.8587	1.2878	1.0854	494.6215	38.9213	213.1781	1.6525			
Del Enthalpy	Del H/U^2	GHP	Reynolds#	Specif Humid	Area4	Area4UB				
242829.766	0.572	648.527	526223.9	0.00187	542.74530	604.79504				
COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 2										
BLEED =	0.000	DPInc =	7.636	EfDer =	0.999	SH =	0.195749E-02			
W act	RPM act	Pt	Tt	POTS	POTH	AeroBl				
47.2600	3620.0000	4.8587	494.6201	1.0000	1.0000	0.9800				
W Kg/sec	Wdry	H2Ovap_Lb/s	H2Ovap_g/m^3	m^3/min	IWAR	IWC				
21.4818	47.1675	0.0925	0.7538	3347.2468	0.0052	2.0000				
W cor	RPM cor	GAMMA	Cp	R	Blades	THK	ArealUB			
139.6273	3707.0652	1.3996	0.2498	53.4118	56	0.0500	602.5374			
CFM	SCFM	Al/A*	Areal	A*	AthrRotor	ChokeMargin	AnnularArea			
118206.898	37053.988	1.4505	590.4866	407.0942	474.1340	1.1647	599.9134			
Station 6. ROTOR LEADING EDGE CONDITIONS, STAGE 2										
TIP	R1	Stator	Alfa	C1	CU1	Cm1	Mabs	Mrel	U1cor	FlowCoeff
20.55	0.00	-0.02	480.45	-0.17	480.45	0.45	0.45	0.76	664.78	0.740
MEAN	18.08	0.00	-0.02	480.45	-0.17	480.45	0.45	0.70		
HUB	15.21	0.00	-0.02	480.45	-0.17	480.45	0.45	0.64		

	BetaFlo	BetaBlade	Incid	U1	W1	Ps1	Ts1	TwetBulb1	RH
TIP	53.50	46.36	7.14	649.18	807.76	4.23	475.44	478.90	29.70
MEAN	49.94	42.30	7.64	571.10	746.44	4.23	475.44	478.90	29.70
HUB	45.01	37.84	7.17	480.49	679.60	4.23	475.44	478.90	29.70

Station 7. ROTOR EXIT CONDITIONS, STAGE 2 SOLUTION IS CONVERGED  
 B2 axial THK AeroBl Blades2 Area2 Area2UB  
 0.5000 0.0500 0.9500 56 541.5062 584.6252

	R2	C2	Cu2	Cm2	Ao2	Mach2	Chord	AxChord	Rcircle
TIP	20.42	562.22	308.79	469.83	1096.90	0.51	3.21	2.81	12.40
MEAN	18.01	574.88	318.06	478.88	1092.38	0.53	3.30	2.93	10.11
HUB	15.22	624.89	389.14	488.94	1087.97	0.57	3.37	3.12	6.23

	U2	W2	Wu2	MachRel2	DelRCu	Eff2uC	Eff2incC	AvgREff	Wsl/W2
TIP	645.08	577.77	336.28	0.53	6308.99	0.92	0.91		
MEAN	568.94	540.62	250.88	0.49	5731.22	0.92	0.91	0.91	1.50
HUB	480.90	497.47	91.76	0.46	5926.34	0.92	0.91		

	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
TIP	5.94	1.22	4.96	526.50	1.06	500.24	495.10	14.56		
MEAN	5.83	1.20	4.83	523.58	1.06	496.12	494.32	16.65	524.88	1.21
HUB	5.87	1.21	4.69	524.56	1.06	492.12	496.12	18.97		

	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter
TIP	33.32	35.59	31.50	4.09	0.93	0.42	1.40	0.00071	25
MEAN	33.59	27.65	23.50	4.15	0.93	0.41	1.63	0.00102	34
HUB	38.52	10.63	6.50	4.13	0.93	0.41	1.97	0.00116	33

Station 8. STATOR ROW #1  
 blockage3 XBladeGap Vane# Area3 Tt3 TwetBulb3 MeltFract Area3UB  
 0.9500 0.7798 73 523.5558 524.8783 495.4347 0.0000 573.6071

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE  
 R3m C3 Cu3 Cm3 Ao3 Mach3 cp 2-3 Stat Ax Chd  
 17.9595 590.7197 318.9518 497.2116 1092.0568 0.5409 -0.0187 2.3644

Pt3 Ps3 Ts3 RH3 FloAlpha3 VaneAlpha3 Incid3  
 5.8684 4.8097 495.9213 17.2172 32.6794 30.6000 2.0794

Station 9. STATOR / VANED DIFFUSER  
 RCG Cth Cuth Cmth Aoth Machth cp 2-Th Stat Chord  
 17.9595 498.3300 105.6495 487.0020 1101.1957 0.4525 0.2478 2.4513

BlockageTh PtTh PsTh TsTh TwetBulbTh AreaTh w2-Th DiffFact4  
 0.9500 5.8557 5.0886 504.2740 493.5365 540.8125 0.0221 0.2612

Station 10. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):  
 R4 C4 Cu4 Cm4 Ao4 Mach4 cp 3-4 Stator Gap  
 17.8235 550.7924 0.0000 550.7924 1096.1786 0.5025 0.0992 0.5911

Blockage4 Ps4 Ts4 TwetBulb4 VaneAlpha4 Vane Thk4 w2-4OD cp 2-4  
 0.9500 4.9148 499.7148 495.4424 0.0000 0.0600 0.0381 0.0824

STAGE EXIT CONDITIONS, STAGE 2  
 Eff4 Pt4 PR TR Tt4 Del T Ns Ns nondim  
 0.8790 5.8389 1.2017 1.0612 524.8765 30.2588 239.0605 1.8532

Del Enthalpy Del H/U^2 GHP Reynolds# Specif Humid Area4 Area4UB  
 189423.453 0.455 505.894 470817.4 0.00267 466.10056 560.16235

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 3  
 BLEED = 0.000 DPInc = 7.726 EfDer = 0.999 SH = 0.275878E-02

W act RPM act Pt Tt POTS POTH AeroBl  
 47.2600 3620.0000 5.8389 524.8750 1.0000 1.0000 0.9800

W Kg/sec Wdry H2Ovap\_Lb/s H2Ovap\_g/m^3 m^3/min IWAR IWC  
 21.4818 47.1296 0.1304 1.2227 2908.1367 0.0045 2.0000

W cor RPM cor GAMMA Cp R Blades THK ArealUB  
 114.8840 3749.1382 1.3984 0.2505 53.4378 56 0.0500 556.6108

CFM SCFM A1/A\* Areal A\* AthrRotor ChokeMargin AnnularArea  
 102699.875 37042.258 1.5623 545.4786 349.1518 429.7651 1.2309 553.8704

Station 11. ROTOR LEADING EDGE CONDITIONS, STAGE 3

	R1	Stator	Alfa	C1	CU1	Cm1	Mabs	Mrel	U1cor	FlowCoeff
TIP	20.07	0.00	-0.02	451.86	-0.16	451.86	0.41	0.70	656.62	0.713
MEAN	17.74	0.00	-0.02	451.86	-0.16	451.86	0.41	0.65		
HUB	15.05	0.00	-0.02	451.86	-0.16	451.86	0.41	0.59		

	BetaFlo	BetaBlade	Incid	U1	W1	Ps1	Ts1	TwetBulb1	RH
TIP	54.53	46.36	8.17	634.02	778.69	5.20	507.96	493.63	13.84
MEAN	51.13	43.40	7.73	560.37	719.97	5.20	507.96	493.63	13.84
HUB	46.47	38.84	7.63	475.44	656.02	5.20	507.96	493.63	13.84

Station 12. ROTOR EXIT CONDITIONS, STAGE 3 SOLUTION IS CONVERGED

B2 axial	THK	AeroBl	Blades2	Area2	Area2UB					
0.5000	0.0500	0.9500	56	502.4182	542.8196					
TIP	R2	C2	Cu2	Cm2	Ao2	Mach2	Chord	AxChord	Rcircle	
MEAN	19.81	540.57	305.66	445.86	1130.10	0.48	3.11	2.72	12.03	
HUB	17.51	551.68	314.49	453.26	1126.08	0.49	3.22	2.84	9.33	
	14.85	598.69	381.37	461.50	1122.14	0.53	3.28	3.03	5.89	
TIP	U2	W2	Wu2	MachRel2	DelRCu	Eff2uC	Eff2incC	AvgREff	Ws1/W2	
MEAN	625.81	548.89	320.15	0.49	6058.29	0.92	0.91	0.91	1.53	
HUB	553.04	512.20	238.55	0.45	5508.44	0.92	0.91	0.91	1.53	
	469.12	469.77	87.75	0.42	5665.63	0.92	0.91	0.91	1.53	
TIP	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
MEAN	7.00	1.20	5.99	555.40	1.06	531.19	507.82	8.45		
HUB	6.89	1.18	5.85	552.63	1.05	527.41	507.12	9.39	553.82	1.19
	6.92	1.19	5.70	553.42	1.05	523.72	508.47	10.41		
TIP	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter	
MEAN	34.43	35.68	31.50	4.18	0.93	0.43	1.40	0.00105	23	
HUB	34.75	27.76	23.50	4.26	0.93	0.42	1.62	0.00134	32	
	39.57	10.77	6.50	4.27	0.93	0.43	1.94	0.00161	30	

Station 13. STATOR ROW #1

blockage3	XBladeGap	Vane#	Area3	Tt3	TwetBulb3	MeltFract	Area3UB			
0.9500	0.7565	73	487.9654	553.8145	508.0614	0.0103	535.3244			
ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE										
R3m	C3	Cu3	Cm3	Ao3	Mach3	cp 2-3	Stat Ax	Chd		
17.3924	564.9846	316.5562	467.9741	1125.9592	0.5018	-0.0128	2.2836			
Pt3	Ps3	Ts3	RH3	FloAlpha3	VaneAlpha3	Incid3				
6.9279	5.8350	527.4150	9.7165	34.0760	31.5000	2.5760				

Station 14. STATOR / VANED DIFFUSER

RCG	Cth	Cuth	Cmth	Aoth	Machth	cp 2-Th	Stat Chord			
17.3924	477.7821	104.2249	466.2755	1133.9329	0.4213	0.2516	2.3726			
BlockageTh	PtTh	PsTh	TsTh	TwetBulbTh	AreaTh	w2-Th	DiffFact4			
0.9500	6.9160	6.1220	534.9377	506.6222	498.3970	0.0209	0.2863			

Station 15. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):

R4	C4	Cu4	Cm4	Ao4	Mach4	cp 3-4	Stator Gap			
17.1214	517.0918	0.0000	517.0918	1130.4536	0.4574	0.1292	0.5709			
Blockage4	Ps4	Ts4	TwetBulb4	VaneAlpha4	Vane Thk4	w2-4OD	cp 2-4			
0.9500	5.9762	531.6927	507.9520	0.0000	0.0600	0.0389	0.1180			

STAGE EXIT CONDITIONS, STAGE 3

Eff4	Pt4	PR	TR	Tt4	Del T	Ns	Ns nondim			
0.8791	6.8964	1.1811	1.0551	553.7888	28.9412	229.7855	1.7813			
Del Enthalpy	Del H/U^2	GHP	Reynolds#	Specif Humid	Area4	Area4UB				
181815.969	0.464	485.577	520427.9	0.00391	434.74258	525.55774				

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 4

BLEED = 0.000 DPInc = 7.916 EfDer = 0.999 SH = 0.402059E-02

W act	RPM act	Pt	Tt	POTS	POTH	AeroBl				
47.2600	3620.0000	6.8964	553.7784	1.0000	1.0000	0.9800				
W Kg/sec	Wdry	H2Ovap_Lb/s	H2Ovap_g/m^3	m^3/min	IWAR	IWC				
21.4818	47.0700	0.1900	2.0215	2563.6169	0.0040	2.0000				
W cor	RPM cor	GAMMA	Cp	R	Blades	THK	Area1UB			
96.9879	3759.9319	1.3967	0.2515	53.4787	56	0.0500	523.8064			



CFM	SCFM	Al/A*	Areal	A*	AthrRotor	ChokeMargin	AnnularArea
90533.273	37023.723	1.6892	513.3303	303.8878	401.0588	1.3198	521.1437

Station 16. ROTOR LEADING EDGE CONDITIONS, STAGE 4

	R1	Stator	Alfa	C1	CU1	Cm1	Mabs	Mrel	U1cor	FlowCoeff
TIP	19.26	0.00	-0.02	423.28	-0.15	423.28	0.37	0.65	631.93	0.696
MEAN	16.97	0.00	-0.02	423.28	-0.15	423.28	0.37	0.60		
HUB	14.32	0.00	-0.02	423.28	-0.15	423.28	0.37	0.54		

	BetaFlo	BetaBlade	Incid	U1	W1	Ps1	Ts1	TwetBulb1	RH
TIP	55.18	46.36	8.82	608.43	741.30	6.27	538.99	506.59	8.18
MEAN	51.72	43.80	7.92	536.11	683.18	6.27	538.99	506.59	8.18
HUB	46.91	37.84	9.07	452.38	619.63	6.27	538.99	506.59	8.18

Station 17. ROTOR EXIT CONDITIONS, STAGE 4 SOLUTION IS CONVERGED

B2 axial	THK	AeroB1	Blades2	Area2	Area2UB				
0.5000	0.0500	0.9500	56	477.1383	516.2928				
	R2	C2	Cu2	Cm2	Ao2	Mach2	Chord	AxChord	Rcircle
TIP	18.88	512.72	294.52	419.69	1160.04	0.44	2.97	2.60	11.47
MEAN	16.57	520.57	299.03	426.11	1156.20	0.45	3.08	2.71	8.74
HUB	13.89	561.06	356.53	433.21	1152.41	0.49	3.11	2.88	5.76

	U2	W2	Wu2	MachRel2	DelRCu	Eff2uC	Eff2incC	AvgREff	Wsl/W2
TIP	596.43	516.99	301.90	0.45	5563.42	0.92	0.91		
MEAN	523.58	481.66	224.54	0.42	4958.60	0.92	0.91	0.91	1.54
HUB	438.79	440.95	82.26	0.38	4954.32	0.92	0.91		

	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
TIP	8.08	1.17	7.07	581.66	1.05	559.96	517.90	5.28		
MEAN	7.94	1.15	6.91	578.63	1.04	556.26	517.08	5.78	579.63	1.16
HUB	7.94	1.15	6.76	578.61	1.04	552.63	517.82	6.32		

	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter
TIP	35.06	35.73	31.50	4.23	0.93	0.44	1.40	0.00119	24
MEAN	35.06	27.79	23.50	4.29	0.93	0.43	1.62	0.00048	32
HUB	39.45	10.75	6.50	4.25	0.93	0.43	1.94	0.00104	28

Station 18. STATOR ROW #1

blockage3	XBladeGap	Vane#	Area3	Tt3	TwetBulb3	MeltFract	Area3UB
0.9500	0.7209	73	476.4520	579.6042	517.5048	0.6762	523.9628

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE

R3m	C3	Cu3	Cm3	Ao3	Mach3	cp 2-3	Stat Ax Chd
16.4812	521.2689	300.7142	425.7842	1157.0745	0.4505	0.0269	2.1803
Pt3	Ps3	Ts3	RH3	FloAlpha3	VaneAlpha3	Incid3	
7.9785	6.9438	557.2090	5.7369	35.2320	32.4000	2.8320	

Station 19. STATOR / VANED DIFFUSER

RCG	Cth	Cuth	Cmth	Aoth	Machth	cp 2-Th	Stat Chord
16.4812	444.4072	99.6675	433.0867	1163.3550	0.3820	0.2711	2.2705
BlockageTh	PtTh	PsTh	TsTh	TwetBulbTh	AreaTh	w2-Th	DiffFact4
0.9500	7.9688	7.2049	563.3000	516.4457	480.4178	0.0190	0.3063

Station 20. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):

R4	C4	Cu4	Cm4	Ao4	Mach4	cp 3-4	Stator Gap
16.0960	470.0504	0.0000	470.0504	1161.2367	0.4048	0.1520	0.5451
Blockage4	Ps4	Ts4	TwetBulb4	VaneAlpha4	Vane Thk4	w2-4OD	cp 2-4
0.9500	7.1010	561.2796	516.9836	0.0000	0.0600	0.0394	0.1746

STAGE EXIT CONDITIONS, STAGE 4

Eff4	Pt4	PR	TR	Tt4	Del T	Ns	Ns nondim
0.8836	7.9469	1.1523	1.0464	579.4802	25.8532	234.1671	1.8152
Del Enthalpy	Del_H/U^2	GHP	Reynolds#	Specif Humid	Area4	Area4UB	
162999.625	0.458	435.324	572371.0	0.00475	425.14447	519.08722	

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 5

BLEED = 0.000 DPInc = 8.019 EfDer = 0.998 SH = 0.493731E-02

W act	RPM act	Pt	Tt	POTS	POTH	AeroB1
47.2600	3620.0000	7.9469	579.4626	1.0000	1.0000	0.9800
W Kg/sec	Wdry	H2Ovap_Lb/s	H2Ovap_g/m^3	m^3/min	IWAR	IWC
21.4818	47.0267	0.2333	2.7747	2293.5586	0.0036	2.0000

W cor	RPM cor	GAMMA	Cp	R	Blades	THK	AreaUB			
83.8205	3775.4858	1.3953	0.2523	53.5084	56	0.0500	520.0693			
CFM	SCFM	Al/A*	Areal	A*	AthrRotor	ChokeMargin	AnnularArea			
80996.250	37010.207	1.8882	509.6679	269.9282	390.2764	1.4459	517.7266			
Station 21. ROTOR LEADING EDGE CONDITIONS, STAGE 5										
	R1	Stator	Alfa	C1	CU1	Cm1	Mabs	Mrel	U1cor	FlowCoeff
TIP	18.32	0.00	-0.02	381.41	-0.13	381.41	0.33	0.59	603.58	0.659
MEAN	15.91	0.00	-0.02	381.41	-0.13	381.41	0.33	0.54		
HUB	13.07	0.00	-0.02	381.41	-0.13	381.41	0.33	0.48		
	BetaFlo	BetaBlade	Incid	U1	W1	Ps1	Ts1	TwetBulb1	RH	
TIP	56.62	47.36	9.26	578.74	693.22	7.38	567.49	516.17	4.90	
MEAN	52.82	44.80	8.02	502.70	631.12	7.38	567.49	516.17	4.90	
HUB	47.28	38.84	8.44	412.89	562.19	7.38	567.49	516.17	4.90	
Station 22. ROTOR EXIT CONDITIONS, STAGE 5 SOLUTION IS CONVERGED										
B2 axial	THK	AeroBl	Blades2	Area2	Area2UB					
0.5000	0.0500	0.9500	56	475.3082	515.3698					
	R2	C2	Cu2	Cm2	Ao2	Mach2	Chord	AxChord	Rcircle	
TIP	17.94	479.28	290.76	381.00	1187.13	0.40	2.82	2.45	10.21	
MEAN	15.50	480.44	284.56	387.11	1182.95	0.41	2.92	2.55	7.91	
HUB	12.59	509.27	323.05	393.69	1178.82	0.43	2.95	2.72	5.29	
	U2	W2	Wu2	MachRel2	DelRCu	Eff2uC	Eff2incC	AvgREff	Ws1/W2	
TIP	566.73	470.45	275.97	0.40	5218.66	0.92	0.91			
MEAN	489.58	438.05	205.02	0.37	4412.04	0.92	0.91	0.91	1.59	
HUB	397.72	400.71	74.68	0.34	4068.90	0.92	0.91			
	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
TIP	9.16	1.15	8.19	605.56	1.05	586.66	527.48	4.12		
MEAN	8.97	1.13	8.01	601.53	1.04	582.54	526.32	4.50	602.30	1.13
HUB	8.88	1.12	7.82	599.81	1.04	578.47	526.29	4.92		
	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter	
TIP	37.35	35.92	31.50	4.42	0.93	0.47	1.40	0.00142	22	
MEAN	36.32	27.91	23.50	4.41	0.93	0.44	1.64	0.00217	29	
HUB	39.37	10.74	6.50	4.24	0.93	0.43	2.01	0.00128	31	
Station 23. STATOR ROW #1										
blockage3	XBladeGap	Vane#	Area3	Tt3	TwetBulb3	MeltFract	Area3UB			
0.9500	0.6802	73	462.6711	602.3022	527.1013	1.0000	510.8013			
ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE										
R3m	C3	Cu3	Cm3	Ao3	Mach3	cp 2-3	Stat Ax Chd			
15.2185	493.5620	289.7749	399.5423	1182.4731	0.4174	-0.0329	2.0437			
Pt3	Ps3	Ts3	RH3	FloAlpha3	VaneAlpha3	Incid3				
8.9842	7.9735	582.3434	4.7215	35.9521	33.0000	2.9521				
Station 24. STATOR / VANED DIFFUSER										
RCG	Cth	Cuth	Cmth	Aoth	Machth	cp 2-Th	Stat Chord			
15.2185	424.4456	96.9225	413.2313	1187.6863	0.3574	0.2092	2.1315			
BlockageTh	PtTh	PsTh	TsTh	TwetBulbTh	AreaTh	w2-Th	DiffFact4			
0.9500	8.9688	8.2143	587.5535	526.4504	461.2568	0.0354	0.3124			
Station 25. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):										
R4	C4	Cu4	Cm4	Ao4	Mach4	cp 3-4	Stator Gap			
14.7013	444.1916	0.0000	444.1916	1186.2103	0.3745	0.1662	0.5109			
Blockage4	Ps4	Ts4	TwetBulb4	VaneAlpha4	Vane Thk4	w2-40D	cp 2-4			
0.9500	8.1415	586.1629	527.8693	0.0000	0.0600	0.0396	0.1363			
STAGE EXIT CONDITIONS, STAGE 5										
Eff4	Pt4	PR	TR	Tt4	Del T	Ns	Ns nondim			
0.8751	8.9646	1.1281	1.0394	602.2983	22.8406	241.9290	1.8754			
Del Enthalpy	Del H/U^2	GHP	Reynolds#	Specif Humid	Area4	Area4UB				
144902.188	0.451	386.991	640835.7	0.00793	410.39981	509.29071				
OVERALL EXIT CONDITIONS; ALL 5 STAGES										
Del Enthal	DelHT/U1^2	GHP	MassFloSlcor	OPR	Efficiency	Rotor1Inc	TR	AxHubLen		
921971.00	2.1707	2462.3127	172.586	2.3760	0.86582	6.6540	1.3217	41.379		

### 7.1.4.2.1 plot.out

The file plot.out is produced from an execution of the COMDES-MELT code. It provides information for each station of the calculation and is mainly used to determine if there is a risk of ice accretion. The variables Twbs and MeltFrac are two of the parameters that determine icing risk.

ZONE	xx	T="M	alt	SH	ThR	RH	g/m3	Twbt	Twbs	Tt	Ts	Pt	Ps	Mach	Enthalpy	MeltFrac
1.0000	40000.0000	0.1395E-02	79.6302	0.4653	453.8118	441.4793	455.7065	442.7897	3.7730	3.4124	0.3814	-40.2053	0.0000			
2.0000	40000.0000	0.1580E-02	49.9181	0.4653	482.7891	451.5374	494.6282	456.4637	4.8597	3.7099	0.6330	44.5418	0.0000			
3.0000	40000.0000	0.1591E-02	50.0076	0.4653	484.0566	451.5749	494.6277	456.5084	4.8950	3.6982	0.6456	2.9166	0.0000			
4.0000	40000.0000	0.1641E-02	30.8983	0.4653	479.2446	460.0702	494.6242	469.6571	4.8752	4.0673	0.5153	21.6704	0.0000			
5.0000	40000.0000	0.1869E-02	33.6638	0.4653	479.9470	461.3481	494.6219	470.7624	4.8587	4.0868	0.5033	14.2352	0.0000			
6.0000	40000.0000	0.1958E-02	29.6973	0.7538	478.9024	464.3555	494.6205	475.4442	4.8587	4.2304	0.4493	8.7184	0.0000			
7.0000	40000.0000	0.2288E-02	16.6485	0.7538	494.3181	476.1749	524.8802	496.1218	5.8320	4.8294	0.5263	50.1516	0.0000			
8.0000	40000.0000	0.2336E-02	17.2167	0.7538	495.4352	476.1867	524.8796	495.9228	5.8684	4.8098	0.5409	3.8885	0.0000			
9.0000	40000.0000	0.2413E-02	13.6024	0.7538	493.5372	480.4937	524.8779	504.2755	5.8557	5.0886	0.4525	10.3667	0.0000			
10.0000	40000.0000	0.2670E-02	17.3257	0.7538	495.4430	478.7365	524.8777	499.7165	5.8389	4.9149	0.5025	1.2757	0.0000			
11.0000	40000.0000	0.2759E-02	13.8356	1.2228	493.6303	482.9136	524.8763	507.9594	6.8909	5.2047	0.4089	9.0612	0.0000			
12.0000	40000.0000	0.3355E-02	9.3942	1.2228	507.1157	492.5249	553.8177	527.4135	6.8909	5.8491	0.4899	53.5052	0.0126			
13.0000	40000.0000	0.3478E-02	9.7161	1.2228	508.0621	492.6884	553.8160	527.4171	6.9280	5.8351	0.5018	10.7746	0.0103			
14.0000	40000.0000	0.3594E-02	8.1613	1.2228	506.6229	496.0190	553.8110	534.9395	6.9161	6.1221	0.4213	31.2993	0.0367			
15.0000	40000.0000	0.3911E-02	9.6692	1.2228	507.9525	495.1137	553.7901	531.6945	6.8965	5.9762	0.4574	130.3384	0.1392			
16.0000	40000.0000	0.4021E-02	8.1780	2.0216	506.5873	498.2862	553.7797	538.9930	6.8965	6.2696	0.3719	65.0653	0.1866			
17.0000	40000.0000	0.4450E-02	5.7796	2.0216	517.0820	505.4290	579.6332	556.2624	7.9445	6.9149	0.4502	721.3027	0.6138			
18.0000	40000.0000	0.4527E-02	5.7368	2.0216	517.5052	505.8555	579.6054	557.2106	7.9785	6.9439	0.4505	173.2100	0.6763			
19.0000	40000.0000	0.4587E-02	5.0235	2.0216	516.4462	508.1948	579.5739	563.3010	7.9689	7.2050	0.3820	195.8688	0.7687			
20.0000	40000.0000	0.4755E-02	5.4500	2.0216	516.9839	507.6157	579.4796	561.2794	7.9470	7.1011	0.4048	587.3492	1.0000			
21.0000	40000.0000	0.4937E-02	4.8976	2.7748	516.1749	510.1423	579.4617	567.4915	7.9470	7.3825	0.3267	111.1108	1.0000			
22.0000	40000.0000	0.6406E-02	4.5013	2.7748	526.3210	516.9411	602.3027	582.5356	8.9664	8.0064	0.4061	105.7949	1.0000			
23.0000	40000.0000	0.6712E-02	4.7218	2.7748	527.1019	517.1427	602.3017	582.3435	8.9843	7.9736	0.4174	6.0845	1.0000			
24.0000	40000.0000	0.7009E-02	4.4083	2.7748	526.4510	519.1953	602.2994	587.5533	8.9689	8.2144	0.3574	14.5129	1.0000			
25.0000	40000.0000	0.7927E-02	5.1308	2.7748	527.8698	519.6741	602.2977	586.1629	8.9648	8.1416	0.3745	10.4327	1.0000			
26.0000	40000.0000	0.9351E-02	5.3700	2.7748	528.3302	523.0033	602.2925	591.9167	8.9648	8.4278	0.2992	32.1823	1.0000			
27.0000	40000.0000	0.1092E-01	6.2791	2.7748	530.0453	524.4763	602.2901	591.8270	8.9648	8.4217	0.3012	15.3438	1.0000			

- xx station number
- alt altitude
- SH  $S_h$
- RH  $\phi$
- g/m3 grams of water vapor per cubic meter of air (not IWC)
- Twbt total  $T_{WB}$
- Twbs  $T_{WBS}$
- Pt  $P_t$
- Ps  $P_s$
- Mach Mach number
- Enthalpy enthalpy—the amount of enthalpy exchanged between the ice particle and the air
- MeltFrac particle melt fraction—ratio of liquid to liquid and ice

The underlined station numbers in the file plot.out listing indicate the region where icing risk conditions exist; that is, Twbs and MeltFrac with a significant IWAR (0.006) as indicated in the comdes.out file. In this case, the risk of ice is in stator 3 (stations 13 to 15) of the LPC.

### 7.1.4.2.2 Melt\_Frac.out (snippet)

The Melt\_Frac.out file is produced during the execution of the MELT subroutine. It provides information about the particle state and the amount of enthalpy extracted from the air and provided to the particle. This data is produced for each blade row, gap, and vane row of a stage.

```
INLET, Station 0.0000000E+00
Pstatic(psia) 14.17000 3.412358
Tstatic(R) 469.5800 442.7870
Velocity(ft/s) 159.4200 393.7570
Distance(in) 0.000000E+00 46.90000
Nsteps 2000
Dist(in) DropTemp(R) MeltFraction
0.023438 469.579987 0.000000
0.046877 469.566589 0.000000
(LINES OF SIMILAR DATA SNIPPED FOR CLARITY)
.
.
46.853039 441.747345 0.000000
46.878476 441.729706 0.000000
46.899914 441.712067 0.000000
ENTHALPY SUM -40.20848
ROTOR Mean, Station 1.000000
Pstatic(psia) 3.412356 3.709634
Tstatic(R) 442.7874 456.4540
Velocity(ft/s) 393.7594 663.4747
Distance(in) 46.90000 54.33419
Nsteps 600
Dist(in) DropTemp(R) MeltFraction
46.912373 441.702240 0.000000
46.924744 441.690521 0.000000
46.937115 441.680084 0.000000
46.949486 441.670807 0.000000
(LINES OF SIMILAR DATA SNIPPED FOR CLARITY)
.
.
Station name and number
Pstatic LE and TE static pressures
Tstatic LE and TE static temperatures
Velocity LE and TE velocities
Distance LE and TE distance from station 1 in inches
Nsteps the number of integration steps taken for integration in the subroutine MELT
Dist distance the particle has traveled (distance)
DropTemp temperature of the particle in Rankine
MeltFraction particle melt fraction—ratio of liquid to liquid and ice
```

spHumidity	$S_h$
RelHumidity	$\phi$
cp	constant pressure specific heat
deltaTemp	change in temperature
cp*deltaTemp*conv	temperature change multiplied by specific heat and a conversion constant
cp*deltaTemp	temperature change multiplied by specific heat
dpm	particle diameter
ENTHALPY SUM	cumulative sum of enthalpy for each component (rotor, gap, stator, etc.)

Care must be taken in interpreting the data in this file as there are currently solutions at the hub, mean, and tip regions of the rotor, which are then passed to the gap and stator at their mean line. This may appear as though there is a discontinuity in particle temperature, melt fraction, etc.

### 7.1.5 Sensitivity of Rotor Exit Conditions to Slip, Blockage, and Efficiency

A sensitivity study was conducted to show the effects of prescribed slip, blockage, and efficiency values on the rotor exit conditions. In this case, the full-fan case was used and executed at its design point while varying the parameters of slip, blockage, and efficiency to understand these effects (Figure 15 to Figure 17).

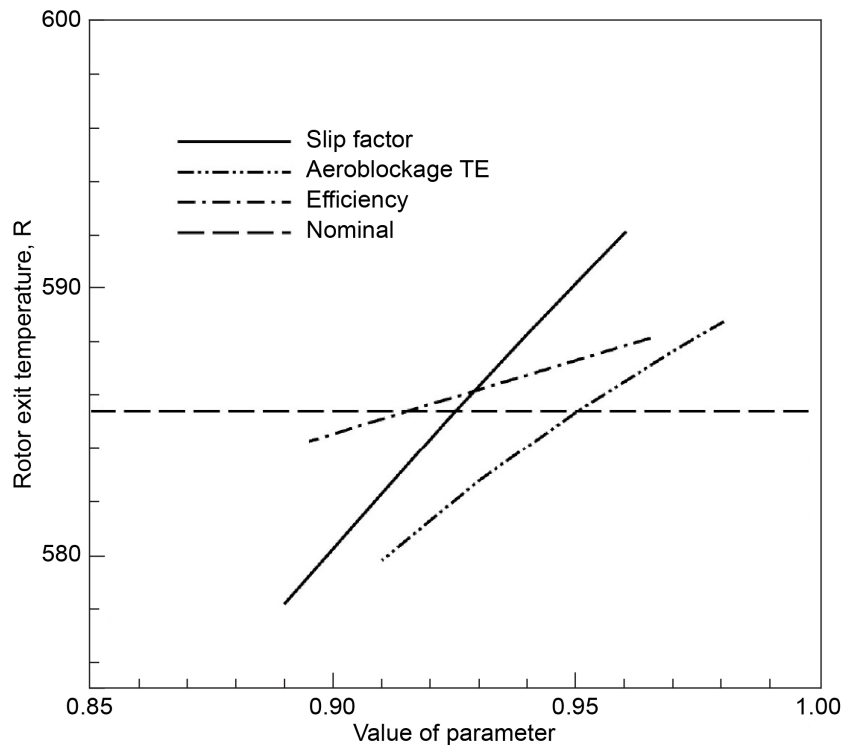


Figure 15.—Sensitivity of rotor exit temperature to slip, blockage, and efficiency.

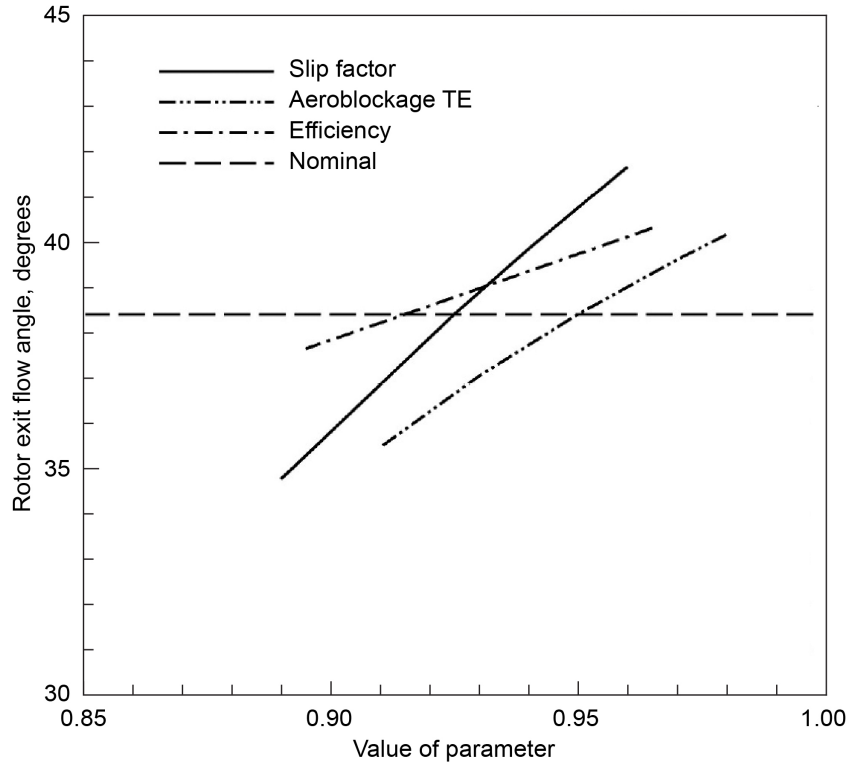


Figure 16.—Sensitivity of rotor exit flow angle to slip, blockage, and efficiency.

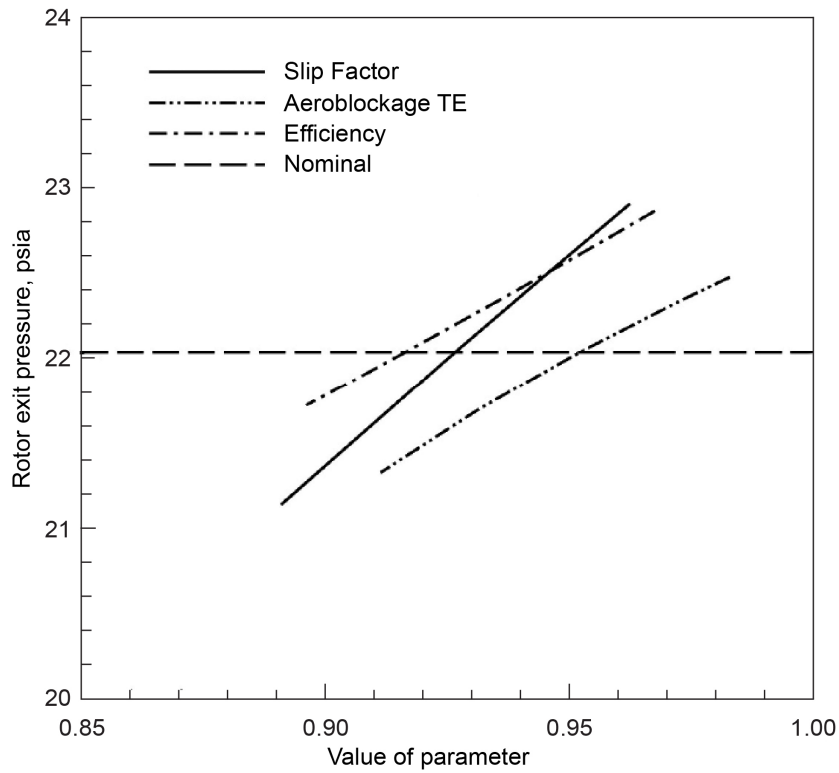


Figure 17.—Sensitivity of rotor exit pressure to slip, blockage, and efficiency.

```

TITLE = "COMDES Parameter Sensitivity"
VARIABLES = Value, Pt3, Tt3, FlowAngle3
Zone T ="Slip", I = 4.0
0.89    21.1405  580.6187  34.7638
0.925   22.0389  587.8586  38.401
0.94    22.4144  590.801   39.8429
0.96    22.9078  594.6     41.6628
Zone T ="AeroB2", I = 5.0
0.98    22.4776  591.2195  40.1589
0.97    22.3382  590.159   39.6068
0.95    22.0389  587.8586  38.401
0.93    21.7061  585.2581  37.0343
0.91    21.3286  582.2524  35.4613
Zone T ="Efficiency", I = 6.0
0.965   22.8663  590.6035  40.3059
0.955   22.6964  590.0615  39.924
0.935   22.3627  588.9634  39.1611
0.915   22.0389  587.8586  38.401
0.905   21.8795  587.2942  38.0211
0.895   21.7236  586.7374  37.6439
Zone T ="Nominal", I = 2.0
0.8     22.0389  587.8586  38.401
1.05   22.0389  587.8586  38.401

```

### 7.1.6 Stage-Matching Example

This stage-matching example is provided to illustrate how a single stage can be the limiting component in the overall performance of the compressor. Figure 18 shows the overall performance of the five-stage compressor. Figure 19 illustrates the performance of stage 1 of the compressor in isolation with the surge and choke limitations of stage 5 indicated. Figure 20 and Figure 21 look at the performance of each stage within the overall compression system. In Figure 21, stage 5 head coefficient is plotted for the rotor only, while the efficiency is for the stage, which includes the stator loss. The high incidence produces a large stator loss, which results in a low efficiency. The rotor produces a positive  $\Delta P_t$  and  $\Delta T_b$ , which gives the rise in enthalpy across the rotor. At low corrected flow rates, the rotor relative velocity ratio,  $W_{s1}/W_2$ , (relative blade tip velocity/relative TE velocity at the mean) (1.82 at station 1 corrected mass flow 136, station 5 corrected mass flow 67) is moving toward surge in stage 5, which limits the range of the compressor. At the higher flow rates, the stage 5 rotor is wind milling.

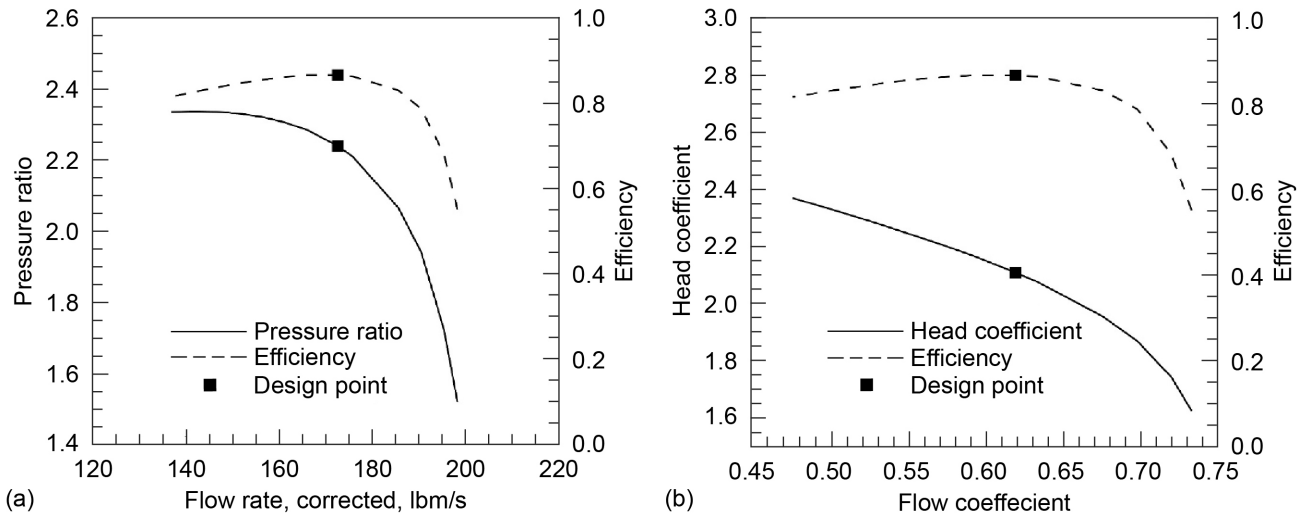


Figure 18.—Overall performance of five-stage compressor with design point indicated (design operating condition coincides with design point for each stage). (a) Pressure ratio and efficiency versus corrected flow rate. (b) Head coefficient and efficiency versus flow coefficient.

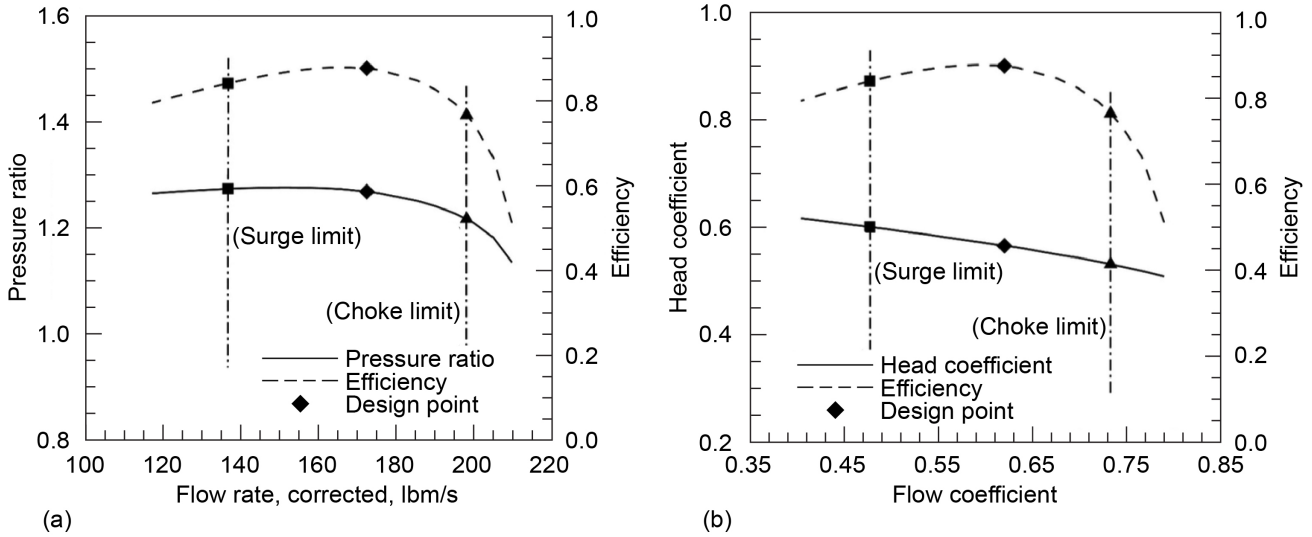


Figure 19.—Stage 1 performance with stage 5 limits for choke (corrected mass flow 198) and surge (corrected mass flow 137) and design operating point. (a) Pressure ratio and efficiency versus corrected flow rate. (b) Head coefficient and efficiency versus flow coefficient.

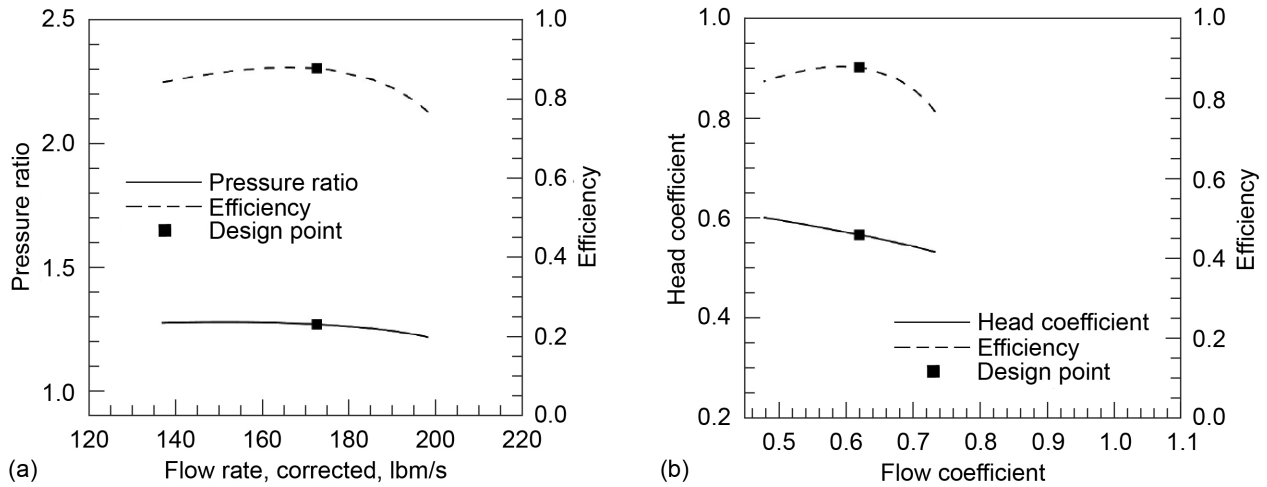
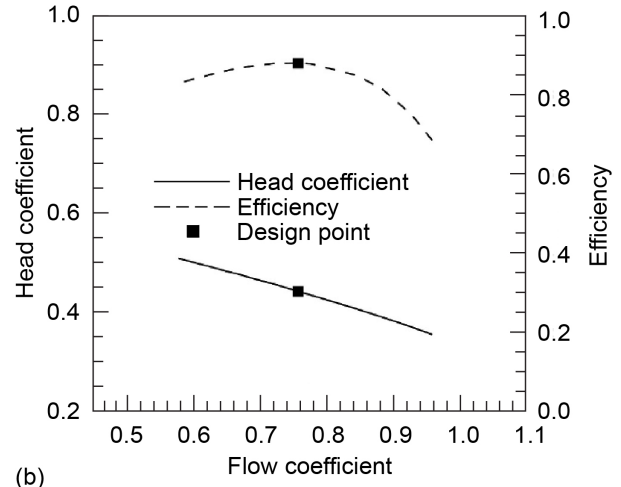
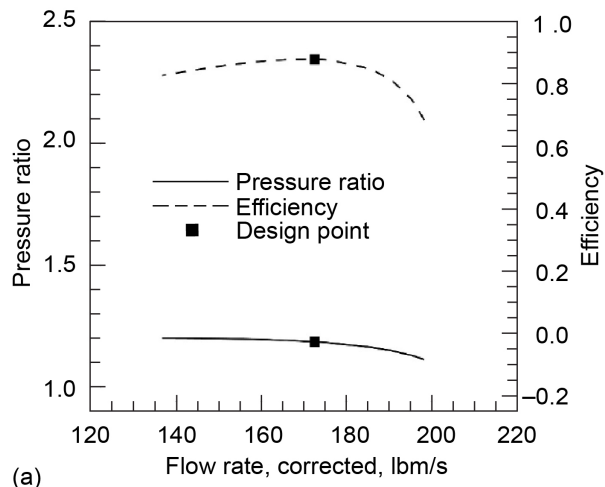


Figure 20.—Stage 1 performance within overall compression system limited by stage 5 surge and choke points, with design operating point indicated. (a) Pressure ratio and efficiency versus corrected flow rate. (b) Head coefficient and efficiency versus flow coefficient. This within system performance of stage 1 can be compared with the isolated stage 1 performance illustrated in Figure 19.

### 7.1.7 40K Turbofan Fan Core and Four-Stage Low-Pressure Compressor Performance Characteristics

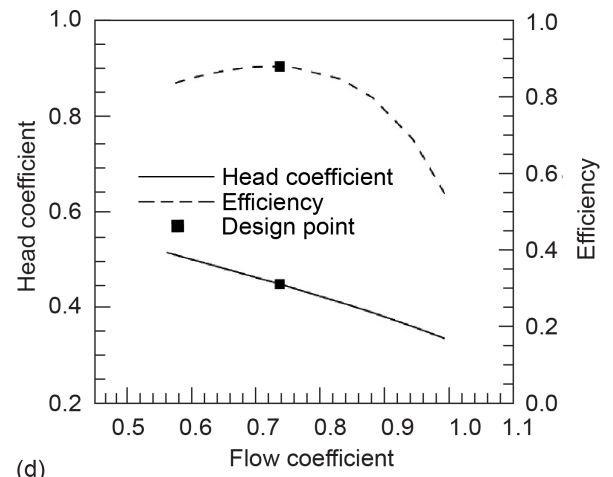
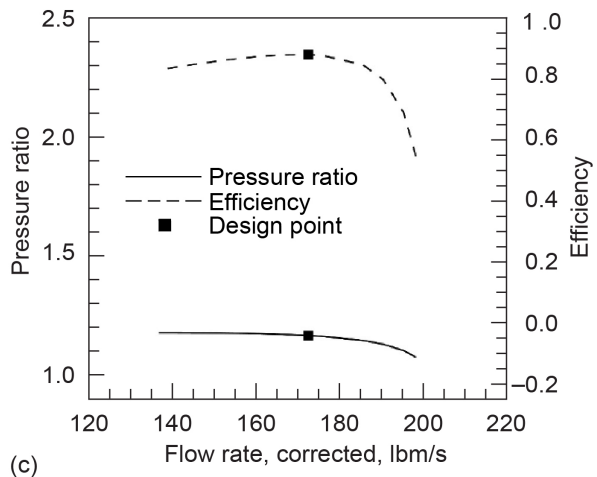
Figure 18 to Figure 21 showcase the pressure ratio and efficiency versus the corrected mass flow and the head coefficient and efficiency versus the flow coefficient.





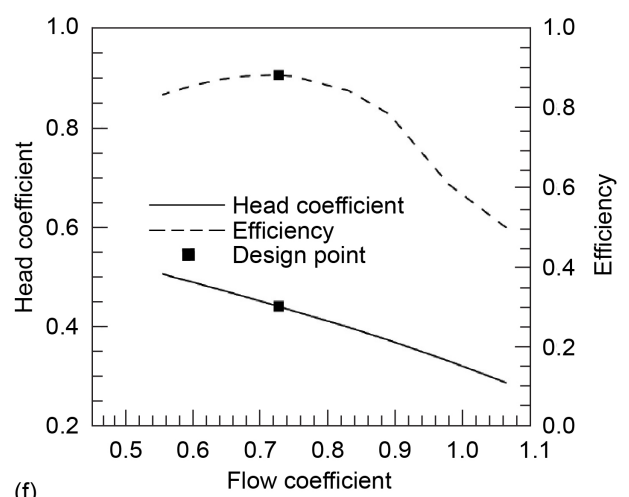
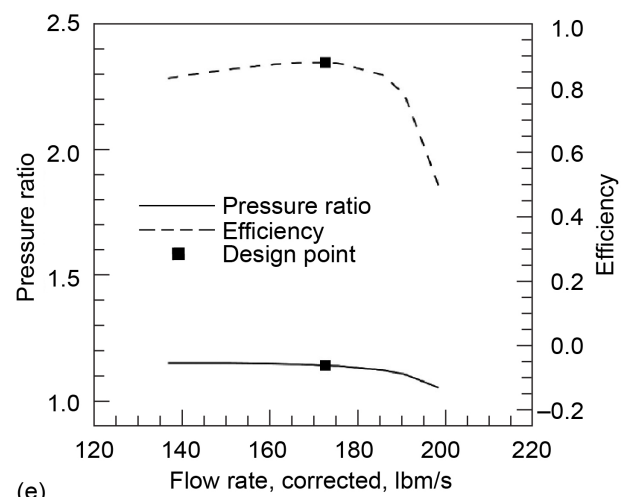
(a)

(b)



(c)

(d)



(e)

(f)

Figure 21.—Stages 2 through 5 performance within the overall compression system limited by the stage 5 surge and choke points, with design operating point indicated. Pressure ratio and efficiency to corrected flow rate. (a) Stage 2, (c) stage 3, (e) stage 4, and (g) stage 5. Head coefficient and efficiency to flow coefficient. (b) Stage 2, (d) stage 3, (f) stage 4, (h) stage 5.

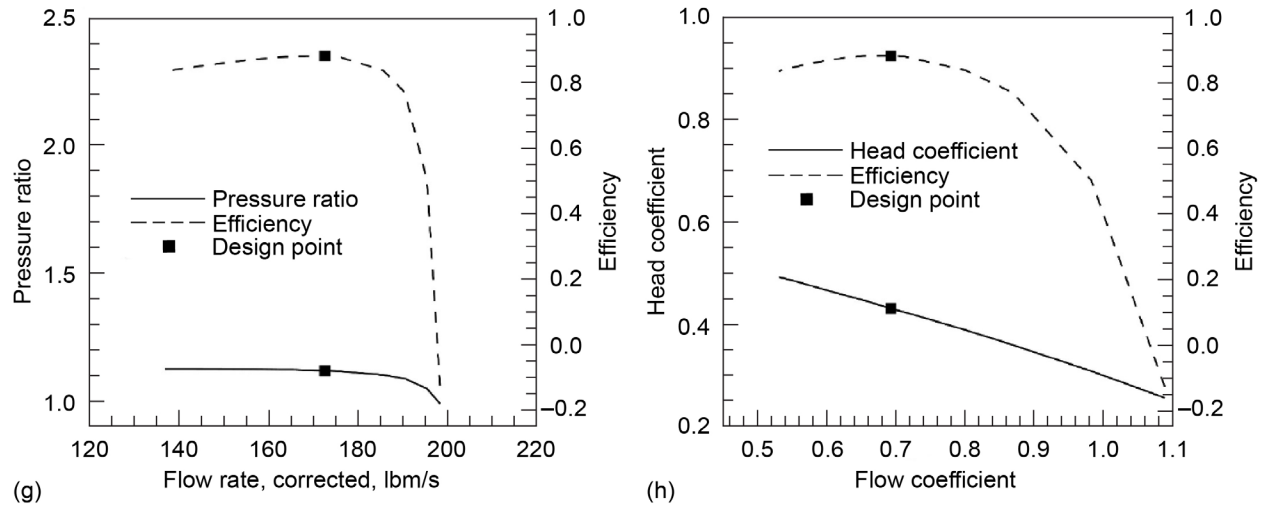


Figure 21.—Concluded.

TABLE IV.—VEHICLE AND ENGINE SYSTEM MODEL

Component	Low-pressure compressor	High-pressure compressor (HPC) option 1 (all-axial four stage)	HPC option 2 (centrifugal stage)
Mass flow, lbm/s	30.0	30.0	30.0
Pressure ratio	13.0	2.73	2.57
Shaft speed, rpm	24,800	32,650	38,620

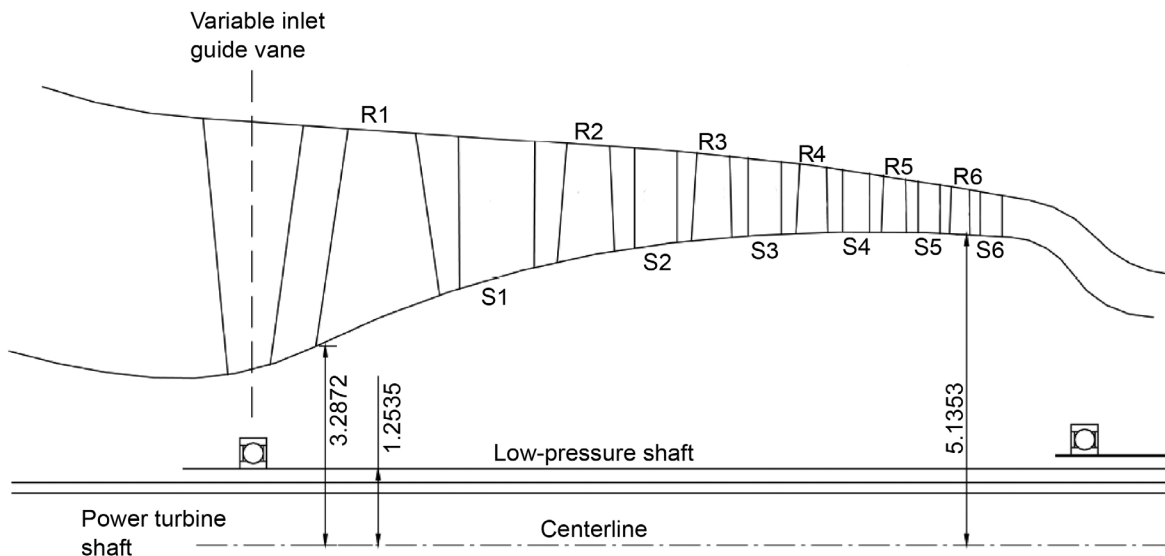


Figure 22.—Flow path for the six-stage axial low-pressure compressor of the Large Civil Transport Rotor Engine (Ref. 12). Pressure ratio = 10.83; efficiency = 84.2 adiabatic; speed = 24,800 RPM; power = 6,400 HP; tip speed = 1,476 ft/s.

## 7.2 Example Case (2): Large Civil Transport Rotor Engine

Table IV is the design table illustrating a vehicle and engine system model. The LPC has 6 axial stages as seen in Figure 22.

## 7.2.1 Input File

comdes.dat

```

SRW 6 Axial Stage LPC 100% RPM 0 IGV 0 S1 0 S2 1-14-2011
30.0      14.7000      518.6700      24800.0      Wact      Pt1      Tt1      RPM
5         6           0           0.00      Fluid     Stages   IDUMMY   DUMMY
0.00     0.0000      00.00      0.00      DropDia   InDist   RHinlet  IWC      ***icing***
14.7     0.00       518.7      0.00      Psin      DUMMY    Tsin     DUMMY    ***icing***
550.0    0.00       00000.00   0         Vin       DUMMY    Altitude IDUMMY   ***icing***
6.900    3.2872      6.8219     4.1321    1 Rltip    Rlhuh    R2tip    R2hub
0.3000   0.3000     21         0.0300    1.0      Blaxial  B2axial  NBLAD    THK      CFS
0.0      0.0000     0.000     1.0000    1.0      StatorT  StatorM  StatorH  FSGV     CFMid
0.9800   0.9500     1.1000    0.9000    1.0      AeroB1   AeroB2   POTS     POTH     CFHub
0.9100   0.910     0.910     0.0000    EfficiT  EfficiM  EfficiH  BLEED
0.0000   0.0000     0.0000    0.0000    DUMMY    DUMMY    DUMMY    DUMMY
0.9200   0.045     1.68      0.00      SlipF    DiffLoss Solidity  RPM2
61.3000  58.15      48.400    0.00      BetaBlade1T BetaBlade1M BetaBlade1H expo3
59.20    48.80      11.70     0         BetaBlade2T BetaBlade2M BetaBlade2H stat2
6.7719   4.2419     0.0        45.6     R3s      R3h      B3X      BetB3
38        0.03       0.95      0.01      Blan3    Blath3   Blok3    Wbar23
00.0     0.000     0.95      0.015     RCG      Athrt    BlokTH   Wbar2T
6.6828   4.5907     0.0        0.0       R4S      R4H      B4X      BetB4
38        0.03       0.95      1.603     Blan4    Blath4   Blok4    SolidStat
6.6459   4.6796     6.5962     4.8635    2 Rltip    Rlhuh    R2tip    R2hub
0.2000   0.2000     32         0.0300    1.0      Blaxial  B2axial  NBLAD    THK      CFS
0.0      0.0000     0.000     1.0000    1.0      StatorT  StatorM  StatorH  FSGV     CFMid
0.9800   0.9500     1.1000    0.9000    1.0      AeroB1   AeroB2   POTS     POTH     CFHub
0.9050   0.905     0.905     0.0000    EfficiT  EfficiM  EfficiH  BLEED
0.0000   0.0000     0.0000    0.0000    DUMMY    DUMMY    DUMMY    DUMMY
0.9200   0.061     1.45      0.00      SlipF    DiffLoss Solidity  RPM2
58.70    57.60      55.30     0.00      BetaBlade1T BetaBlade1M BetaBlade1H expo3
55.2     48.8       35.00     0         BetaBlade2T BetaBlade2M BetaBlade2H stat2
6.5671   4.9158     0.0        47.3     R3s      R3h      B3X      BetB3
66        0.03       0.95      0.01      Blan3    Blath3   Blok3    Wbar23
00.0     0.000     0.95      0.015     RCG      Athrt    BlokTH   Wbar2T
6.5163   5.0054     0.0        0.0       R4S      R4H      B4X      BetB4
66        0.03       0.95      1.50      Blan4    Blath4   Blok4    SolidStat
6.4878   5.0305     6.4360     5.0914    3 Rltip    Rlhuh    R2tip    R2hub
0.1500   0.1500     36         0.0300    1.0      Blaxial  B2axial  NBLAD    THK      CFS
0.0      0.0000     0.000     1.0000    1.0      StatorT  StatorM  StatorH  FSGV     CFMid
0.9800   0.9500     1.1000    0.9000    1.0      AeroB1   AeroB2   POTS     POTH     CFHub
0.9000   0.900     0.900     0.0000    EfficiT  EfficiM  EfficiH  BLEED
0.0000   0.0000     0.0000    0.0000    DUMMY    DUMMY    DUMMY    DUMMY
0.9200   0.067     1.38      0.00      SlipF    DiffLoss Solidity  RPM2
60.3000  60.00      59.200    0.00      BetaBlade1T BetaBlade1M BetaBlade1H expo3
56.50    52.10      43.60     0         BetaBlade2T BetaBlade2M BetaBlade2H stat2
6.4045   5.1110     0.0        48.5     R3s      R3h      B3X      BetB3
79        0.03       0.95      0.01      Blan3    Blath3   Blok3    Wbar23
00.0     0.000     0.95      0.015     RCG      Athrt    BlokTH   Wbar2T
19.411   14.474     0.0        0.0       R4S      R4H      B4X      BetB4
79        0.03       0.95      1.40      Blan4    Blath4   Blok4    SolidStat
6.3053   5.1553     6.2478     5.1727    4 Rltip    Rlhuh    R2tip    R2hub
0.1000   0.1000     44         0.0300    1.0      Blaxial  B2axial  NBLAD    THK      CFS
0.0      0.0000     0.000     1.0000    1.0      StatorT  StatorM  StatorH  FSGV     CFMid
0.9800   0.9500     1.1000    0.9000    1.0      AeroB1   AeroB2   POTS     POTH     CFHub
0.8950   0.895     0.895     0.0000    EfficiT  EfficiM  EfficiH  BLEED
0.0000   0.0000     0.0000    0.0000    DUMMY    DUMMY    DUMMY    DUMMY
0.9200   0.071     1.30      0.00      SlipF    DiffLoss Solidity  RPM2
60.700   60.8       60.700    0.00      BetaBlade1T BetaBlade1M BetaBlade1H expo3
56.60    53.30      46.70     0         BetaBlade2T BetaBlade2M BetaBlade2H stat2
6.2111   5.1777     0.0        48.7     R3s      R3h      B3X      BetB3
96        0.03       0.95      0.01      Blan3    Blath3   Blok3    Wbar23
00.0     0.000     0.95      0.015     RCG      Athrt    BlokTH   Wbar2T
18.485   13.284     0.0        0.0       R4S      R4H      B4X      BetB4
96        0.03       0.95      1.361     Blan4    Blath4   Blok4    SolidStat
6.1113   5.1742     6.0534     5.168     5 Rltip    Rlhuh    R2tip    R2hub
0.1000   0.100     51         0.0300    1.0      Blaxial  B2axial  NBLAD    THK      CFS
0.0      0.0000     0.000     1.0000    1.0      StatorT  StatorM  StatorH  FSGV     CFMid
0.9800   0.9500     1.1000    0.9000    1.0      AeroB1   AeroB2   POTS     POTH     CFHub
0.8900   0.890     0.890     0.0000    EfficiT  EfficiM  EfficiH  BLEED
0.0000   0.0000     0.0000    0.0000    DUMMY    DUMMY    DUMMY    DUMMY
0.9200   0.073     1.270     0.00      SlipF    DiffLoss Solidity  RPM2
60.4000  60.90      61.2000    0.00      BetaBlade1T BetaBlade1M BetaBlade1H expo3

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55.50	52.30	47.40	0	BetaBlade2T	BetaBlade2M	BetaBlade2H	stat2			
6.0198	5.1647	0.0	45.0	R3s	R3h	B3X	BetB3			
115	0.06	0.95	0.02	Blan3	Blath3	Blok3	Wbar23			
6.0	23.000	0.95	0.03	RCG	Athrt	BlokTH	Wbar2T			
17.239	11.622	0.0	0.0	R4S	R4H	B4X	BetB4			
115	0.03	0.95	1.374	Blan4	Blath4	Blok4	SolidStat			
5.9245	5.1525	5.8729	5.1353	6 R1tip	Rlhub	R2tip	R2hub			
0.1000	0.100	60	0.0300	1.0	Blaxial	B2axial	NBLAD	THK	CFS	
0.0	0.0000	0.000	1.0000	1.0	StatorT	StatorM	StatorH	FSGV	CFMid	
0.9800	0.9500	1.1000	0.9000	1.0	AeroB1	AeroB2	POTS	POTH	CFHub	
0.8850	0.885	0.885	0.0000	EfficiT	EfficiM	EfficiH	BLEED			
0.0000	0.0000	0.0000	0.0000	DUMMY	DUMMY	DUMMY	DUMMY			
0.9200	0.057	1.26	0.00	SlipF	DiffLoss	Solidity	RPM2			
58.1000	58.50	59.3000	0.00	BetaBlade1T	BetaBlade1M	BetaBlade1H	expo3			
53.00	50.60	46.50	0	BetaBlade2T	BetaBlade2M	BetaBlade2H	stat2			
5.8430	5.1473	0.0	43.0	R3s	R3h	B3X	BetB3			
123	0.06	0.95	0.02	Blan3	Blath3	Blok3	Wbar23			
6.0	23.000	0.95	0.03	RCG	Athrt	BlokTH	Wbar2T			
5.7797	5.0958	0.0	0.0	R4S	R4H	B4X	BetB4			
123	0.03	0.95	1.474	Blan4	Blath4	Blok4	SolidStat			
5.8430	5.1473	0.00	2.0	R7S	R7h	B7X	Length67			
4	0.01	0.98	0.000	Blan7	Blath7	Blok7	Wbar67			
5.8430	5.1473	0.0	2.0	R8S	R8h	B8X	Length78			
4	0.01	0.98	0.00	Blan8	Blath8	Blok8	Wbar78			

## 7.2.2 Output File

### comdes.out

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*****
***** AXIAL & CENTRIFUGAL COMPRESSOR BLADE DESIGN 1-D ****
***** COMDES-MELT Release Version 1 *****
***** Gasplus Fluid Properties *****
*****

SRW 6 Axial Stage LPC 100% RPM 0 IGV 0 S1 0 S2 1-14-2011

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 1
BLEED = 0.000 DPinc = 5.997 EfDer = 0.999 SH = 0.000000E+00

W act      RPM act      Pt          Tt          POTS        POTH        AeroB1
30.0000    24800.0000    14.6999    518.6700    1.1000      0.9000      0.9800

W Kg/sec   Wdry          H2Ovap_Lb/s H2Ovap_g/m^3 m^3/min     IWAR        IWC
13.6364    30.0000      0.0000     0.0000     760.4573    0.0000      0.0000

W cor      RPM cor      GAMMA       Cp          R           Blades      THK        ArealUB
29.9991    24800.7188    1.4015     0.2486     53.3483     21          0.0300     116.0221

CFM        SCFM        A1/A*       Areal       A*          AthrRotor   ChokeMargin AnnularArea
26855.293  23539.482    1.3013     113.7017    87.3738     64.9162     0.7430     115.6242

Station 1. ROTOR LEADING EDGE CONDITIONS, STAGE 1
R1         Stator     Alfa       Cl          CU1         Cm1         Mabs       Mrel       Ulcor      FlowCoeff
TIP        6.90      0.00      -0.02      623.54     -0.21      623.54     0.58      1.50      1493.30    0.380
MEAN       5.40      0.00      -0.02      566.86     -0.20      566.86     0.52      1.20
HUB        3.29      0.00      -0.02      510.17     -0.18      510.17     0.47      0.80

BetaFlo    BetaBlade  Incid     U1          W1          Ps1         Ts1        TwetBulb1 RH
TIP        67.34     61.30     6.04      1493.30    1618.46    11.73      486.22     495.69     0.00
MEAN       64.15     58.15     6.00      1169.63    1299.93    12.21      491.86     495.69     0.00
HUB        54.36     48.40     5.96      711.42     875.58     12.66      496.95     495.69     0.00

Station 2. ROTOR EXIT CONDITIONS, STAGE 1 SOLUTION IS CONVERGED
B2 axial   THK        AeroB1     Blades2     Area2       Area2UB
0.3000     0.0300     0.9500     21          86.8613     93.1381

R2         C2         Cu2        Cm2         Ao2         Mach2      Chord     AxChord    Rcircle
TIP        6.82      743.29    581.36     463.14     1211.88    0.61      3.43      1.91      93.56
MEAN       5.64      740.58    515.15     532.05     1175.03    0.63      3.18      2.00      19.49
HUB        4.13      918.14    699.60     594.60     1149.06    0.80      2.46      2.13      3.90

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	U2	W2	Wu2	MachRel2	DelRCu	Eff2uC	Eff2incC	AvgREff	Ws1/W2
TIP	1476.40	1007.77	895.04	0.83	3967.48	0.91	0.91		
MEAN	1220.55	883.56	705.40	0.75	2906.35	0.91	0.91	0.91	1.93
HUB	894.27	625.66	194.67	0.54	2891.40	0.91	0.91		

	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
TIP	31.32	2.13	24.29	656.65	1.27	610.54	559.02	0.00		
MEAN	25.98	1.77	19.88	619.74	1.19	573.98	544.71	0.00	631.87	1.89
HUB	25.91	1.76	17.01	619.22	1.19	548.88	544.50	0.00		

	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter
TIP	51.46	62.64	59.20	3.44	0.92	0.48	1.68	0.00007	115
MEAN	44.08	52.97	48.80	4.17	0.92	0.42	1.96	0.00018	60
HUB	49.64	18.13	11.70	6.43	0.92	0.46	2.50	0.00049	55

Station 3. STATOR ROW #1

blockage3	XBladeGap	Vane#	Area3	Tt3	TwetBulb3	MeltFract	Area3UB
0.9500	0.5317	38	80.4232	631.8702	549.5632	0.0000	87.5402

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE

R3m	C3	Cu3	Cm3	Ao3	Mach3	cp 2-3	Stat Ax Chd
5.6503	754.5968	514.1817	552.2985	1185.5018	0.6365	0.1605	1.6547

Pt3	Ps3	Ts3	RH3	FloAlpha3	VaneAlpha3	Incid3
27.6631	21.0563	584.3815	0.0000	42.9531	45.6000	-2.6469

Station 4. STATOR / VANED DIFFUSER

RCG	Cth	Cuth	Cmth	Aoth	Machth	cp 2-Th	Stat Chord
5.6503	719.9306	225.3369	683.7568	1189.6989	0.6051	0.1513	1.7949

BlockageTh	PtTh	PsTh	TsTh	TwetBulbTh	AreaTh	w2-Th	DiffFact4
0.9500	27.5412	21.5027	588.6710	549.4588	67.4523	0.0266	0.2999

Station 5. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):

R4	C4	Cu4	Cm4	Ao4	Mach4	cp 3-4	Stator Gap
5.7330	689.7365	0.0000	689.7365	1193.2628	0.5780	0.1002	0.4137

Blockage4	Ps4	Ts4	TwetBulb4	VaneAlpha4	Vane Thk4	w2-4OD	cp 2-4
0.9500	21.7184	592.2217	549.1984	0.0000	0.0300	0.0677	0.1806

STAGE EXIT CONDITIONS, STAGE 1

Eff4	Pt4	PR	TR	Tt4	Del T	Ns	Ns nondim
0.8843	27.2394	1.8530	1.2183	631.8702	113.2003	291.3328	2.2584

Del Enthalpy	Del_H/U^2	GHP	Reynolds#	Specif Humid	Area4	Area4UB
704975.188	0.323	1195.164	1046373.7	0.00000	63.27254	74.09536

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 2  
 BLEED = 0.000 DPInc = 6.655 EfDer = 1.000 SH = 0.000000E+00

W act	RPM act	Pt	Tt	POTS	POTH	AeroBl
30.0000	24800.0000	27.2394	631.8702	1.1000	0.9000	0.9800

W Kg/sec	Wdry	H2Ovap_Lb/s	H2Ovap_g/m^3	m^3/min	IWAR	IWC
13.6364	30.0000	0.0000	0.0000	494.5772	0.0000	0.0000

W cor	RPM cor	GAMMA	Cp	R	Blades	THK	ArealUB
17.8689	22469.6250	1.4005	0.2490	53.3483	32	0.0300	70.3221

CFM	SCFM	A1/A*	Area1	A*	AthrRotor	ChokeMargin	AnnularArea
17465.828	23539.482	1.3421	68.9157	51.3502	39.4607	0.7685	69.9612

Station 6. ROTOR LEADING EDGE CONDITIONS, STAGE 2

	R1	Stator	Alfa	C1	CU1	Cm1	Mabs	Mrel	U1cor	FlowCoeff
TIP	6.65	0.00	-0.02	659.94	-0.23	659.94	0.55	1.32	1303.12	0.417
MEAN	5.75	0.00	-0.02	599.95	-0.21	599.95	0.50	1.15		
HUB	4.68	0.00	-0.02	539.95	-0.19	539.95	0.45	0.95		

	BetaFlo	BetaBlade	Incid	U1	W1	Ps1	Ts1	TwetBulb1	RH
TIP	65.36	58.70	6.66	1438.31	1582.69	22.15	595.58	549.20	0.00
MEAN	64.25	57.60	6.65	1243.87	1381.18	22.98	601.90	549.20	0.00
HUB	61.94	55.30	6.64	1012.76	1147.87	23.75	607.58	549.20	0.00

Station 7. ROTOR EXIT CONDITIONS, STAGE 2 SOLUTION IS CONVERGED

B2 axial	THK	AeroBl	Blades2	Area2	Area2UB					
0.2000	0.0300	0.9500	32	58.0639	62.7943					
	R2	C2	Cu2	Cm2	Ao2	Mach2	Chord	AxChord	Rcircle	
TIP	6.60	781.37	609.02	489.52	1316.09	0.59	1.88	1.15	30.75	
MEAN	5.79	762.25	544.50	533.42	1290.75	0.59	1.90	1.20	12.38	
HUB	4.86	805.25	570.28	568.52	1273.05	0.63	1.82	1.28	5.15	
	U2	W2	Wu2	MachRel2	DelRCu	Eff2uC	Eff2incC	AvgREff	Wsl/W2	
TIP	1427.56	953.75	818.54	0.72	4018.73	0.90	0.90			
MEAN	1254.15	887.77	709.65	0.69	3156.57	0.90	0.90	0.90	1.84	
HUB	1052.56	745.53	482.29	0.59	2774.41	0.90	0.90			
	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
TIP	51.50	1.89	40.57	771.39	1.22	720.52	596.83	0.00		
MEAN	45.35	1.66	35.81	741.46	1.17	693.05	587.48	0.00	747.01	1.71
HUB	42.80	1.57	32.69	728.19	1.15	674.17	583.21	0.00		
	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter	
TIP	51.21	59.12	55.20	3.92	0.92	0.53	1.45	0.00011	90	
MEAN	45.59	53.07	48.80	4.27	0.92	0.47	1.68	0.00016	53	
HUB	45.09	40.31	35.00	5.31	0.92	0.48	1.98	0.00021	39	

Station 8. STATOR ROW #1

blockage3	XBladeGap	Vane#	Area3	Tt3	TwetBulb3	MeltFract	Area3UB		
0.9500	0.3201	66	53.4854	747.0098	589.2399	0.0000	59.5700		

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE

R3m	C3	Cu3	Cm3	Ao3	Mach3	cp 2-3	Stat Ax Chd		
5.8005	793.1229	543.9836	577.1705	1291.8364	0.6139	0.0200	0.8590		
Pt3	Ps3	Ts3	RH3	FloAlpha3	VaneAlpha3	Incid3			
46.4455	36.0161	694.6949	0.0000	43.3045	47.3000	-3.9955			

Station 9. STATOR / VANED DIFFUSER

RCG	Cth	Cuth	Cmth	Aoth	Machth	cp 2-Th	Stat Chord		
5.8005	837.9829	271.7140	792.7086	1286.0309	0.6516	-0.1539	0.9378		
BlockageTh	PtTh	PsTh	TsTh	TwetBulbTh	AreaTh	w2-Th	DiffFact4		
0.9500	46.2655	34.7881	688.6486	589.1183	42.0700	0.0277	0.3394		

Station 10. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):

R4	C4	Cu4	Cm4	Ao4	Mach4	cp 3-4	Stator Gap		
5.8102	705.9301	0.0000	705.9301	1301.8038	0.5423	0.1115	0.2148		
Blockage4	Ps4	Ts4	TwetBulb4	VaneAlpha4	Vane Thk4	w2-4OD	cp 2-4		
0.9500	37.1787	705.5840	588.5323	0.0000	0.0300	0.1119	0.0807		

STAGE EXIT CONDITIONS, STAGE 2

Eff4	Pt4	PR	TR	Tt4	Del T	Ns	Ns nondim		
0.8610	45.4068	1.6670	1.1822	747.0098	115.1396	229.8738	1.7820		

Del Enthalpy Del\_H/U^2 GHP Reynolds# Specif Humid Area4 Area4UB

719174.500	0.353	1219.236	791600.3	0.00000	43.02640	54.68928			
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COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 3

BLEED = 0.000 DPInc = 6.597 EfDer = 1.000 SH = 0.000000E+00

W act	RPM act	Pt	Tt	POTS	POTH	AeroBl			
30.0000	24800.0000	45.4068	747.0098	1.1000	0.9000	0.9800			
W Kg/sec	Wdry	H2Ovap_Lb/s	H2Ovap_g/m^3	m^3/min	IWAR	IWC			
13.6364	30.0000	0.0000	0.0000	337.8088	0.0000	0.0000			
W cor	RPM cor	GAMMA	Cp	R	Blades	THK	Area1UB		
10.5598	22809.4609	1.3986	0.2500	53.3483	36	0.0300	53.0122		
CFM	SCFM	Al/A*	Area1	A*	AthrRotor	ChokeMargin	AnnularArea		
11929.604	23539.482	1.5503	51.9519	33.5114	27.5109	0.8209	52.7336		

Station 11. ROTOR LEADING EDGE CONDITIONS, STAGE 3

	R1	Stator	Alfa	C1	CU1	Cm1	Mabs	Mrel	U1cor	FlowCoeff
TIP	6.49	0.00	-0.02	598.22	-0.21	598.22	0.46	1.16	1291.36	0.387
MEAN	5.81	0.00	-0.02	543.84	-0.19	543.84	0.41	1.04		
HUB	5.03	0.00	-0.02	489.45	-0.17	489.45	0.37	0.90		

	BetaFlo	BetaBlade	Incid	U1	W1	Ps1	Ts1	TwetBulb1	RH
TIP	66.93	60.30	6.63	1404.10	1526.41	39.38	717.30	588.53	0.00
MEAN	66.60	60.00	6.60	1256.34	1369.16	40.38	722.41	588.53	0.00
HUB	65.80	59.20	6.60	1088.71	1193.82	41.30	727.11	588.53	0.00

Station 12. ROTOR EXIT CONDITIONS, STAGE 3 SOLUTION IS CONVERGED

B2 axial	THK	AeroBl	Blades2	Area2	Area2UB					
0.1500	0.0300	0.9500	36	45.1580	48.9959					

	R2	C2	Cu2	Cm2	Ao2	Mach2	Chord	AxChord	Rcircle
TIP	6.44	737.47	564.52	474.53	1409.47	0.52	1.55	0.91	23.38
MEAN	5.80	716.20	511.76	501.04	1391.99	0.51	1.62	0.95	11.73
HUB	5.09	734.40	517.75	520.85	1380.09	0.53	1.63	1.02	6.00

	U2	W2	Wu2	MachRel2	DelRCu	Eff2uC	Eff2incC	AvgREff	Wsl/W2
TIP	1392.88	954.65	828.36	0.68	3634.61	0.90	0.90		
MEAN	1255.84	897.05	744.08	0.64	2970.71	0.90	0.90	0.90	1.74
HUB	1101.89	782.62	584.14	0.57	2636.90	0.90	0.90		

	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
TIP	74.47	1.64	61.81	872.72	1.17	827.57	625.23	0.00		
MEAN	68.38	1.51	57.10	849.76	1.14	807.18	618.85	0.00	853.56	1.53
HUB	65.46	1.44	53.99	838.21	1.12	793.44	615.60	0.00		

	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter
TIP	49.95	60.19	56.50	3.69	0.92	0.51	1.38	0.00017	67
MEAN	45.61	56.04	52.10	3.94	0.92	0.46	1.60	0.00011	44
HUB	44.83	48.28	43.60	4.68	0.92	0.46	1.85	0.00039	32

Station 13. STATOR ROW #1

blockage3	XBladeGap	Vane#	Area3	Tt3	TwetBulb3	MeltFract	Area3UB
0.9500	0.2538	79	41.5429	853.5625	619.8912	0.0000	46.7950

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE

R3m	C3	Cu3	Cm3	Ao3	Mach3	cp 2-3	Stat Ax Chd
5.7940	748.9904	512.5384	546.1603	1391.1523	0.5384	-0.0137	0.6502

Pt3	Ps3	Ts3	RH3	FloAlpha3	VaneAlpha3	Incid3
69.3203	56.9333	807.1265	0.0000	43.1810	48.5000	-5.3190

Station 14. STATOR / VANED DIFFUSER

RCG	Cth	Cuth	Cmth	Aoth	Machth	cp 2-Th	Stat Chord
5.7940	842.0095	279.6828	794.2024	1380.4034	0.6100	-0.3247	0.7131

BlockageTh	PtTh	PsTh	TsTh	TwetBulbTh	AreaTh	w2-Th	DiffFact4
0.9500	69.1240	53.7976	794.9160	619.7861	31.3433	0.0266	1.0635

Station 15. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):

R4	C4	Cu4	Cm4	Ao4	Mach4	cp 3-4	Stator Gap
17.1214	43.2751	0.0000	43.2751	1430.4662	0.0303	0.8908	0.1626

Blockage4	Ps4	Ts4	TwetBulb4	VaneAlpha4	Vane Thk4	w2-4OD	cp 2-4
0.9500	67.9682	853.4075	619.1855	0.0000	0.0300	0.1208	0.8755

STAGE EXIT CONDITIONS, STAGE 3

Eff4	Pt4	PR	TR	Tt4	Del T	Ns	Ns nondim
0.8500	68.0117	1.4978	1.1426	853.5625	106.5527	200.6499	1.5554

Del Enthalpy	Del_H/U^2	GHP	Reynolds#	Specif Humid	Area4	Area4UB
668847.812	0.345	1133.916	933604.1	0.00000	464.35901	525.55774

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 4

BLEED = 0.000 DPInc = 6.565 EfDer = 1.000 SH = 0.000000E+00

W act	RPM act	Pt	Tt	POTS	POTH	AeroBl
30.0000	24800.0000	68.0117	853.5625	1.1000	0.9000	0.9800

W Kg/sec	Wdry	H2Ovap_Lb/s	H2Ovap_g/m^3	m^3/min	IWAR	IWC				
13.6364	30.0000	0.0000	0.0000	253.3617	0.0000	0.0000				
W cor	RPM cor	GAMMA	Cp	R	Blades	THK	ArealUB			
6.9310	23201.1641	1.3958	0.2513	53.3483	44	0.0300	41.5615			
CFM	SCFM	Al/A*	Areal	A*	AthrRotor	ChokeMargin	AnnularArea			
8947.385	23539.482	1.7045	40.7302	23.8958	20.7314	0.8676	41.4052			

Station 16. ROTOR LEADING EDGE CONDITIONS, STAGE 4

	R1	Stator	Alfa	C1	CU1	Cm1	Mabs	Mrel	U1cor	FlowCoeff
TIP	6.31	0.00	-0.02	571.76	-0.20	571.76	0.41	1.05	1276.59	0.381
MEAN	5.76	0.00	-0.02	519.79	-0.18	519.79	0.37	0.96		
HUB	5.16	0.00	-0.02	467.81	-0.16	467.81	0.33	0.86		
	BetaFlo	BetaBlade	Incid	U1	W1	Ps1	Ts1	TwetBulb1	RH	
TIP	67.27	60.70	6.57	1364.60	1479.72	60.72	826.55	619.19	0.00	
MEAN	67.37	60.80	6.57	1246.38	1350.59	61.94	831.13	619.19	0.00	
HUB	67.26	60.70	6.56	1115.71	1209.97	63.07	835.48	619.19	0.00	

Station 17. ROTOR EXIT CONDITIONS, STAGE 4 SOLUTION IS CONVERGED

B2 axial	THK	AeroBl	Blades2	Area2	Area2UB					
0.1000	0.0300	0.9500	44	35.4486	38.7395					
	R2	C2	Cu2	Cm2	Ao2	Mach2	Chord	AxChord	Rcircle	
TIP	6.25	701.88	505.49	486.94	1485.87	0.47	1.16	0.68	16.21	
MEAN	5.74	686.16	464.82	504.74	1473.62	0.47	1.23	0.71	9.42	
HUB	5.17	706.35	483.23	515.19	1467.06	0.48	1.27	0.75	5.23	
	U2	W2	Wu2	MachRel2	DelRCu	Eff2uC	Eff2incC	AvgREff	Ws1/W2	
TIP	1352.15	976.70	846.66	0.66	3159.46	0.89	0.89			
MEAN	1241.28	926.09	776.46	0.63	2667.01	0.89	0.89	0.89	1.63	
HUB	1119.48	818.68	636.25	0.56	2500.41	0.89	0.89			
	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
TIP	99.51	1.46	85.45	962.26	1.13	921.57	648.10	0.00		
MEAN	94.03	1.38	81.08	945.32	1.11	906.43	643.73	0.00	949.05	1.40
HUB	92.22	1.36	78.73	939.58	1.10	898.38	642.24	0.00		
	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter	
TIP	46.07	60.10	56.60	3.50	0.92	0.47	1.30	0.00003	50	
MEAN	42.64	56.97	53.30	3.67	0.92	0.43	1.50	0.00013	36	
HUB	43.17	51.00	46.70	4.30	0.92	0.44	1.73	0.00033	28	

Station 18. STATOR ROW #1

blockage3	XBladeGap	Vane#	Area3	Tt3	TwetBulb3	MeltFract	Area3UB			
0.9500	0.1886	96	32.2979	949.0532	644.6548	0.0000	36.9740			

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE

R3m	C3	Cu3	Cm3	Ao3	Mach3	cp 2-3	Stat Ax Chd			
5.7178	726.2781	466.2602	556.8494	1471.8896	0.4934	-0.0361	0.5041			
Pt3	Ps3	Ts3	RH3	FloAlpha3	VaneAlpha3	Incid3				
95.1180	80.5915	905.6486	0.0000	39.9401	48.7000	-8.7599				

Station 19. STATOR / VANED DIFFUSER

RCG	Cth	Cuth	Cmth	Aoth	Machth	cp 2-Th	Stat Chord			
5.7178	915.2496	305.2154	862.8588	1450.8912	0.6308	-0.6777	0.5533			
BlockageTh	PtTh	PsTh	TsTh	TwetBulbTh	AreaTh	w2-Th	DiffFact4			
0.9500	94.8139	72.6016	880.1555	644.5212	23.6696	0.0325	1.0684			

Station 20. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):

R4	C4	Cu4	Cm4	Ao4	Mach4	cp 3-4	Stator Gap			
16.0960	36.5041	0.0000	36.5041	1507.0150	0.0242	0.8390	0.1260			
Blockage4	Ps4	Ts4	TwetBulb4	VaneAlpha4	Vane Thk4	w2-4OD	cp 2-4			
0.9500	92.7786	948.9434	643.6342	0.0000	0.0300	0.1805	0.8167			

STAGE EXIT CONDITIONS, STAGE 4

Eff4	Pt4	PR	TR	Tt4	Del T	Ns	Ns nondim			
0.8177	92.8166	1.3647	1.1119	949.0532	95.4907	187.6902	1.4550			



Del Enthalpy Del\_H/U^2 GHP Reynolds# Specif Humid Area4 Area4UB  
 603167.625 0.330 1022.566 1118423.0 0.00000 448.42819 519.08722

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 5  
 BLEED = 0.000 DPInc = 6.005 EfDer = 0.999 SH = 0.000000E+00

W act RPM act Pt Tt POTS POTH AeroBl  
 30.0000 24800.0000 92.8166 949.0532 1.1000 0.9000 0.9800

W Kg/sec Wdry H2Ovap\_Lb/s H2Ovap\_g/m^3 m^3/min IWAR IWC  
 13.6364 30.0000 0.0000 0.0000 205.1698 0.0000 0.0000

W cor RPM cor GAMMA Cp R Blades THK Area1UB  
 5.0099 23519.9648 1.3925 0.2529 53.3483 51 0.0300 33.4130

CFM SCFM Al/A\* Areal A\* AthrRotor ChokeMargin AnnularArea  
 7245.503 23539.482 1.7764 32.7447 18.4332 16.3940 0.8894 33.2244

Station 21. ROTOR LEADING EDGE CONDITIONS, STAGE 5  
 R1 Stator Alfa Cl CU1 Cm1 Mabs Mrel Ulcor FlowCoeff  
 TIP 6.11 0.00 -0.02 574.91 -0.20 574.91 0.39 0.97 1254.31 0.395  
 MEAN 5.66 0.00 -0.02 522.64 -0.18 522.64 0.35 0.89  
 HUB 5.17 0.00 -0.02 470.38 -0.16 470.38 0.32 0.81

BetaFlo BetaBlade Incid U1 W1 Ps1 Ts1 TwetBulb1 RH  
 TIP 66.51 60.40 6.11 1322.61 1442.34 83.74 921.91 643.63 0.00  
 MEAN 66.90 60.90 6.00 1225.41 1332.38 85.27 926.42 643.63 0.00  
 HUB 67.22 61.20 6.02 1119.80 1214.74 86.67 930.87 643.63 0.00

Station 22. ROTOR EXIT CONDITIONS, STAGE 5 SOLUTION IS CONVERGED  
 B2 axial THK AeroBl Blades2 Area2 Area2UB  
 0.1000 0.0300 0.9500 51 28.5458 31.4115

R2 C2 Cu2 Cm2 Ao2 Mach2 Chord AxChord Rcircle  
 TIP 6.05 688.17 458.18 513.46 1549.90 0.44 0.95 0.57 11.08  
 MEAN 5.63 685.92 442.37 524.21 1542.59 0.44 1.01 0.59 6.77  
 HUB 5.17 697.11 449.86 532.53 1537.22 0.45 1.08 0.63 4.48

U2 W2 Wu2 MachRel2 DelRCu Eff2uC Eff2incC AvgREff Ws1/W2  
 TIP 1310.08 994.67 851.90 0.64 2774.78 0.89 0.89  
 MEAN 1218.05 936.20 775.68 0.61 2490.72 0.89 0.89 0.89 1.55  
 HUB 1118.46 854.76 668.60 0.56 2325.73 0.89 0.89

Pt2 PR Ps2 Tt2 TR Ts2 TwetBulb2 RH Tt2avg PR2avg  
 TIP 125.56 1.35 109.74 1043.92 1.10 1005.04 667.33 0.00  
 MEAN 121.88 1.31 106.48 1034.21 1.09 995.58 664.95 0.00 1035.57 1.32  
 HUB 119.77 1.29 104.09 1028.57 1.08 988.67 663.57 0.00

Alfa2 Beta FLO Beta BLADE Deviat Slip F. DiffFct Solidity Convergence Iter  
 TIP 41.74 58.92 55.50 3.42 0.92 0.43 1.27 0.00009 37  
 MEAN 40.16 55.95 52.30 3.65 0.92 0.41 1.45 0.00037 31  
 HUB 40.19 51.46 47.40 4.06 0.92 0.41 1.69 0.00012 24

Station 23. STATOR ROW #1  
 blockage3 XBladeGap Vane# Area3 Tt3 TwetBulb3 MeltFract Area3UB  
 0.9500 0.1571 115 22.9383 1035.5664 665.1730 0.0000 30.0458

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE  
 R3m C3 Cu3 Cm3 Ao3 Mach3 cp 2-3 Stat Ax Chd  
 5.6086 810.7252 443.9102 678.3943 1530.6191 0.5297 -0.3519 0.4175

Pt3 Ps3 Ts3 RH3 FloAlpha3 VaneAlpha3 Incid3  
 122.0923 100.9795 981.8271 0.0000 33.1989 45.0000 -11.8011

Station 24. STATOR / VANED DIFFUSER  
 RCG Cth Cuth Cmth Aoth Machth cp 2-Th Stat Chord  
 6.0000 737.6679 227.9519 701.5639 1537.9580 0.4796 -0.2110 0.4519

BlockageTh PtTh PsTh TsTh TwetBulbTh AreaTh w2-Th DiffFact4  
 0.9500 120.9950 103.4712 991.0514 664.7609 23.0000 0.0902 1.0631

Station 25. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):  
 R4 C4 Cu4 Cm4 Ao4 Mach4 cp 3-4 Stator Gap  
 14.7013 32.4792 0.0000 32.4792 1571.8368 0.0207 0.8904 0.1044

Blockage4 Ps4 Ts4 TwetBulb4 VaneAlpha4 Vane Thk4 w2-4OD cp 2-4  
 0.9500 119.7785 1035.4801 664.3136 0.0000 0.0300 0.1657 0.8320

STAGE EXIT CONDITIONS, STAGE 5  
 Eff4 Pt4 PR TR Tt4 Del T Ns Ns nondim  
 0.8118 119.8140 1.2909 1.0912 1035.5664 86.5132 180.8001 1.4016

Del Enthalpy Del\_H/U^2 GHP Reynolds# Specif Humid Area4 Area4UB  
 550172.500 0.321 932.722 1310604.4 0.00000 425.99030 509.29071

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 6  
 BLEED = 0.000 DPInc = 7.171 EfDer = 1.000 SH = 0.000000E+00

W act RPM act Pt Tt POTS POTH AeroBl  
 30.0000 24800.0000 119.8140 1035.5664 1.1000 0.9000 0.9800

W Kg/sec Wdry H2Ovap\_Lb/s H2Ovap\_g/m^3 m^3/min IWAR IWC  
 13.6364 30.0000 0.0000 0.0000 173.3370 0.0000 0.0000

W cor RPM cor GAMMA Cp R Blades THK ArealUB  
 3.8447 23742.1777 1.3890 0.2546 53.3483 60 0.0300 27.0896

CFM SCFM Al/A\* Areal A\* AthrRotor ChokeMargin AnnularArea  
 6121.339 23539.482 1.7836 26.5478 14.8841 14.1345 0.9496 26.8652

Station 26. ROTOR LEADING EDGE CONDITIONS, STAGE 6

	R1	Stator	Alfa	C1	CU1	Cm1	Mabs	Mrel	U1cor	FlowCoeff
TIP	5.92	0.00	-0.02	597.67	-0.21	597.67	0.39	0.91	1227.46	0.424
MEAN	5.55	0.00	-0.02	543.34	-0.19	543.34	0.35	0.85		
HUB	5.15	0.00	-0.02	489.01	-0.17	489.01	0.31	0.78		

	BetaFlo	BetaBlade	Incid	U1	W1	Ps1	Ts1	TwetBulb1	RH
TIP	65.01	58.10	6.91	1282.19	1414.83	108.20	1006.41	664.31	0.00
MEAN	65.67	58.50	7.17	1201.55	1318.86	110.16	1011.17	664.31	0.00
HUB	66.32	59.30	7.02	1115.11	1217.77	111.95	1016.04	664.31	0.00

Station 27. ROTOR EXIT CONDITIONS, STAGE 6 SOLUTION IS CONVERGED

B2 axial	THK	AeroBl	Blades2	Area2	Area2UB
0.1000	0.0300	0.9500	60	23.1821	25.7420

	R2	C2	Cu2	Cm2	Ao2	Mach2	Chord	AxChord	Rcircle
TIP	5.87	704.05	440.39	549.31	1606.87	0.44	0.77	0.49	8.71
MEAN	5.52	697.92	417.52	559.26	1600.24	0.44	0.84	0.51	6.10
HUB	5.14	708.30	426.86	565.23	1596.53	0.44	0.91	0.55	4.07

	U2	W2	Wu2	MachRel2	DelRCu	Eff2uC	Eff2incC	AvgREff	Wsl/W2
TIP	1271.02	995.83	830.63	0.62	2587.58	0.88	0.88		
MEAN	1193.87	956.82	776.35	0.60	2304.25	0.88	0.88	0.88	1.49
HUB	1111.39	887.73	684.53	0.56	2192.92	0.88	0.88		

	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
TIP	155.15	1.29	136.11	1123.44	1.08	1083.00	685.28	0.00		
MEAN	150.96	1.26	132.59	1113.82	1.08	1074.08	683.02	0.00	1115.76	1.27
HUB	149.33	1.25	130.58	1110.03	1.07	1069.10	682.12	0.00		

	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter
TIP	38.72	56.52	53.00	3.52	0.92	0.42	1.26	0.00037	36
MEAN	36.74	54.23	50.60	3.63	0.92	0.38	1.44	0.00037	31
HUB	37.06	50.45	46.50	3.95	0.92	0.37	1.68	0.00040	24

Station 28. STATOR ROW #1

blockage3	XBladeGap	Vane#	Area3	Tt3	TwetBulb3	MeltFract	Area3UB
0.9500	0.1370	123	17.9419	1115.7618	683.3568	0.0000	24.0205

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE

R3m	C3	Cu3	Cm3	Ao3	Mach3	cp 2-3	Stat Ax Chd
5.5061	867.8878	418.2986	760.4312	1584.2594	0.5478	-0.4746	0.4093

Pt3	Ps3	Ts3	RH3	FloAlpha3	VaneAlpha3	Incid3
151.4371	123.7077	1054.5884	0.0000	28.8143	43.0000	-14.1857

```

Station 29. STATOR / VANED DIFFUSER
RCG      Cth      Cuth      Cmth      Aoth      Machth      cp 2-Th      Stat Chord
6.0000   614.0399   181.5765   586.5790   1607.4213   0.3820      0.1284      0.4400

BlockageTh PtTh      PsTh      TsTh      TwetBulbTh AreaTh      w2-Th      DiffFact4
0.9500   149.7212   135.4982   1085.1000   682.7960   23.0000      0.1117      0.0232

Station 30. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):
R4      C4      Cu4      Cm4      Ao4      Mach4      cp 3-4      Stator Gap
5.4485   990.4385   0.0000   990.4385   1569.9607   0.6309      -0.3441      0.1023

Blockage4 Ps4      Ts4      TwetBulb4 VaneAlpha4 Vane Thk4 w2-40D      cp 2-4
0.9500   114.1660   1036.2054   682.5295   0.0000      0.0300      0.1549      -1.0114

STAGE EXIT CONDITIONS, STAGE      6
Eff4      Pt4      PR      TR      Tt4      Del T      Ns      Ns nondim
0.8094   148.9117   1.2429   1.0774   1115.7618   80.1954   174.7721   1.3548

Del Enthalpy Del_H/U^2 GHP      Reynolds#      Specif Humid Area4      Area4UB
513613.938   0.318   870.744   1496132.5   0.00000   14.66639   23.36639

OVERALL EXIT CONDITIONS; ALL      6 STAGES
Del Enthal DelHT/U1^2 GHP      MassFloSlcor OPR      Efficiency      RotorlInc TR      AxHubLen
3759951.50   1.6861   6374.3481   29.999   10.1300   0.80239   5.9967   2.1512   13.560

```

### 7.3 Example Case (3): Large Civil Transport Rotor Engine

The input and output files in this section are those used for the analysis of option 1 for the HPC consisting of four-stage axial compressor for this example case.

#### 7.3.1 High-Pressure Compressor: Option 1 Four Axial Stages

Figure 23 illustrates the four-stage HPC configuration for the Large Civil Transport Rotor Engine.

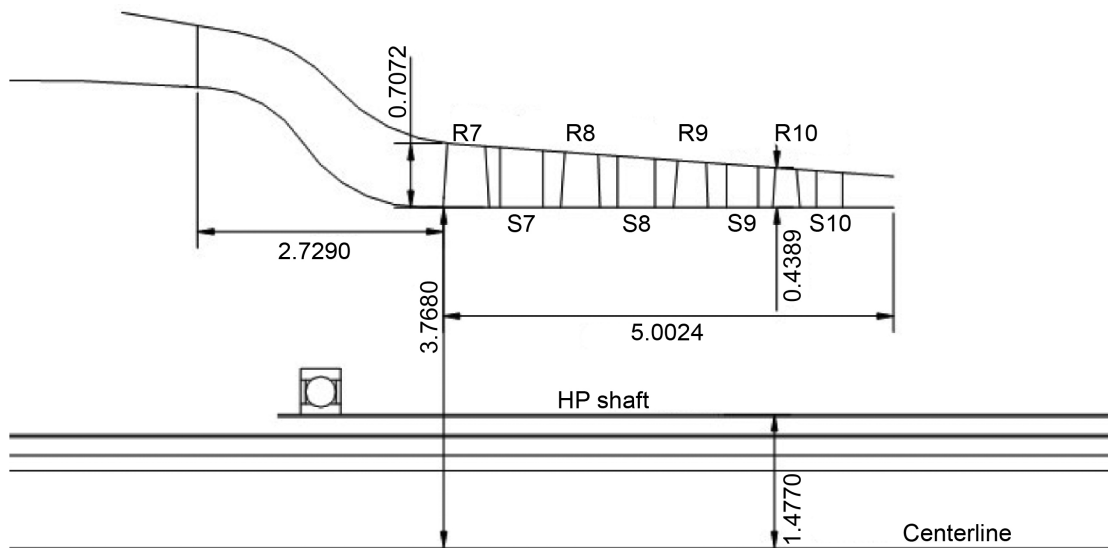


Figure 23.—Flow path for the four-stage axial high-pressure (HP) compressor of the Large Civil Transport Rotor Engine, option 1 (Ref. 12).

### 7.3.1.1 Input File

comdes.dat

Four Axials HPC 100% SRW 1-21-2011				Wact	Pt1	Tt1	RPM		
30.0	159.25	1119.39	32650.0	Fluid	Stages	IDUMMY	DUMMY		
5	4	0	0.00	DropDia	InDist	RHinlet	IWC	***icing***	
0.00	46.900	000.00	0.00	Psin	DUMMY	Tsin	DUMMY	***icing***	
2.70	0.00	406.44	0.00	Vin	DUMMY	Altitude	IDUMMY	***icing***	
607.8	0.00	00000.00	0	1 Rltip	Rlhuh	R2tip	R2hub		
4.4752	3.768	4.4442	3.7680	1.0	Blaxial	B2axial	NBLAD	THK	CFS
0.1000	0.100	59	0.0300	1.0	StatorT	StatorM	StatorH	FSGV	CFMid
0.0	0.0000	0.000	1.0000	1.0	AeroB1	AeroB2	POTS	POTH	CFHub
0.9800	0.9500	1.1000	0.9000	EfficiT	EfficiM	EfficiH	BLEED		
0.920	0.920	0.920	0.0000	DUMMY	DUMMY	DUMMY	DUMMY		
0.00	0.00	0.00	0.00	SlipF	DiffLoss	Solidity	RPM2		
0.9270	0.0629	1.50	0.00	BetaBlade1T	BetaBlade1M	BetaBlade1H	expo3		
53.2000	54.10	54.5000	0.00	BetaBlade2T	BetaBlade2M	BetaBlade2H	stat2		
40.30	35.60	27.30	0	R3s	R3h	B3X	BetB3		
4.4387	3.768	0.0	35.5	Blan3	Blath3	Blok3	Wbar23		
74	0.03	0.95	0.01	RCG	Athrt	BlokTH	Wbar2T		
00.0	0.000	0.95	0.015	R4S	R4H	B4X	BetB4		
4.4102	3.768	0.0	0.0	Blan4	Blath4	Blok4	SolidStat		
74	0.03	0.95	1.30	2 Rltip	Rlhuh	R2tip	R2hub		
4.3791	3.768	4.3519	3.7680	0.9	Blaxial	B2axial	NBLAD	THK	CFS
0.1000	0.100	60	0.0300	0.9	StatorT	StatorM	StatorH	FSGV	CFMid
0.0	0.0000	0.000	1.0000	0.9	AeroB1	AeroB2	POTS	POTH	CFHub
0.9800	0.9500	1.1000	0.9000	EfficiT	EfficiM	EfficiH	BLEED		
0.915	0.915	0.915	0.0000	DUMMY	DUMMY	DUMMY	DUMMY		
0.00	0.00	0.00	0.00	SlipF	DiffLoss	Solidity	RPM2		
0.9270	0.0663	1.450	0.00	BetaBlade1T	BetaBlade1M	BetaBlade1H	expo3		
54.7000	55.60	56.1000	0.00	BetaBlade2T	BetaBlade2M	BetaBlade2H	stat2		
41.20	37.00	30.70	0	R3s	R3h	B3X	BetB3		
4.4354	3.768	0.0	38.5	Blan3	Blath3	Blok3	Wbar23		
81	0.03	0.95	0.01	RCG	Athrt	BlokTH	Wbar2T		
00.0	0.000	0.95	0.015	R4S	R4H	B4X	BetB4		
4.3189	3.768	0.0	0.0	Blan4	Blath4	Blok4	SolidStat		
81	0.03	0.95	1.30	3 Rltip	Rlhuh	R2tip	R2hub		
4.2868	3.768	4.2636	3.7680	0.9	Blaxial	B2axial	NBLAD	THK	CFS
0.1000	0.100	68	0.0300	0.9	StatorT	StatorM	StatorH	FSGV	CFMid
0.0	0.0000	0.000	1.0000	0.9	AeroB1	AeroB2	POTS	POTH	CFHub
0.9800	0.9500	1.1000	0.9000	EfficiT	EfficiM	EfficiH	BLEED		
0.915	0.915	0.915	0.0000	DUMMY	DUMMY	DUMMY	DUMMY		
0.00	0.00	0.00	0.00	SlipF	DiffLoss	Solidity	RPM2		
0.9270	0.0712	1.450	0.00	BetaBlade1T	BetaBlade1M	BetaBlade1H	expo3		
54.9000	55.80	56.4000	0.00	BetaBlade2T	BetaBlade2M	BetaBlade2H	stat2		
40.20	36.50	30.70	0	R3s	R3h	B3X	BetB3		
4.2677	3.768	0.0	30.5	Blan3	Blath3	Blok3	Wbar23		
94	0.03	0.95	0.01	RCG	Athrt	BlokTH	Wbar2T		
00.0	0.000	0.95	0.015	R4S	R4H	B4X	BetB4		
4.2396	3.768	0.0	0.0	Blan4	Blath4	Blok4	SolidStat		
94	0.03	0.95	1.30	4 Rltip	Rlhuh	R2tip	R2hub		
4.2069	3.768	4.1896	3.7680	0.9	Blaxial	B2axial	NBLAD	THK	CFS
0.1000	0.100	80	0.0300	0.9	StatorT	StatorM	StatorH	FSGV	CFMid
0.0	0.0000	0.000	1.0000	0.9	AeroB1	AeroB2	POTS	POTH	CFHub
0.9800	0.9500	1.1000	0.9000	EfficiT	EfficiM	EfficiH	BLEED		
0.915	0.915	0.915	0.0000	DUMMY	DUMMY	DUMMY	DUMMY		
0.00	0.00	0.00	0.00	SlipF	DiffLoss	Solidity	RPM2		
0.9270	0.0753	1.450	0.00	BetaBlade1T	BetaBlade1M	BetaBlade1H	expo3		
54.7000	55.60	56.5000	0.00	BetaBlade2T	BetaBlade2M	BetaBlade2H	stat2		
38.70	35.00	29.90	0	R3s	R3h	B3X	BetB3		
4.2022	3.768	0.0	30.0	Blan3	Blath3	Blok3	Wbar23		
104	0.03	0.95	0.01	RCG	Athrt	BlokTH	Wbar2T		
00.0	0.000	0.95	0.015	R4S	R4H	B4X	BetB4		
4.1978	3.768	0.0	0.0	Blan4	Blath4	Blok4	SolidStat		
104	0.03	0.95	1.30	R7S	R7h	B7X	Length67		
4.1978	3.768	0.00	2.0	Blan7	Blath7	Blok7	Wbar67		
4	0.01	0.98	0.000	R8S	R8h	B8X	Length78		
4.1978	3.768	0.0	2.0	Blan8	Blath8	Blok8	Wbar78		
4	0.01	0.98	0.00						

### 7.3.1.2 Output File

comdes.out

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*****
***** AXIAL & CENTRIFUGAL COMPRESSOR BLADE DESIGN 1-D *****
***** COMDES-MELT Release Version 1 *****
***** Gasplus Fluid Properties *****
*****

Four Axials HPC 100% SRW 1-21-2011

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 1
BLEED = 0.000 DPInc = 6.015 EfDer = 0.999 SH = 0.000000E+00

W act RPM act Pt Tt POTS POTH AeroBl
30.0000 32650.0000 159.2484 1119.3900 1.1000 0.9000 0.9800

W Kg/sec Wdry H2Ovap_Lb/s H2Ovap_g/m^3 m^3/min IWAR IWC
13.6364 30.0000 0.0000 0.0000 144.8933 0.0000 0.0000

W cor RPM cor GAMMA Cp R Blades THK ArealUB
4.0681 22225.4531 1.3855 0.2563 53.3483 59 0.0300 18.4964

CFM SCFM Al/A* Areal A* AthrRotor ChokeMargin AnnularArea
5116.858 23539.482 1.5240 18.1265 11.8942 10.6401 0.8946 18.3142

Station 1. ROTOR LEADING EDGE CONDITIONS, STAGE 1

R1 Stator Alfa C1 CU1 Cm1 Mabs Mrel U1cor FlowCoeff
TIP 4.48 0.00 -0.02 745.24 -0.26 745.24 0.47 0.92 867.96 0.531
MEAN 4.14 0.00 -0.02 677.49 -0.23 677.49 0.42 0.85
HUB 3.77 0.00 -0.02 609.74 -0.21 609.74 0.38 0.77

BetaFlo BetaBlade Incid U1 W1 Ps1 Ts1 TwetBulb1 RH
TIP 59.70 53.20 6.50 1275.10 1477.13 137.41 1074.34 686.19 0.00
MEAN 60.11 54.10 6.01 1178.66 1359.70 141.03 1082.16 686.19 0.00
HUB 60.41 54.50 5.91 1073.60 1234.85 144.37 1089.21 686.19 0.00

Station 2. ROTOR EXIT CONDITIONS, STAGE 1 SOLUTION IS CONVERGED
B2 axial THK AeroBl Blades2 Area2 Area2UB
0.1000 0.0300 0.9500 59 15.6041 17.6353

R2 C2 Cu2 Cm2 Ao2 Mach2 Chord AxChord Rcircle
TIP 4.44 900.42 620.38 652.60 1672.45 0.54 0.71 0.55 3.16
MEAN 4.12 904.53 610.11 667.78 1664.31 0.54 0.76 0.57 2.37
HUB 3.77 936.84 644.11 680.30 1658.10 0.57 0.80 0.61 1.71

U2 W2 Wu2 MachRel2 DelRCu Eff2uC Eff2incC AvgREff Ws1/W2
TIP 1266.26 918.18 645.88 0.55 2758.24 0.92 0.92
MEAN 1173.89 873.94 563.78 0.53 2514.63 0.92 0.92 0.92 1.71
HUB 1073.60 804.53 429.49 0.49 2427.79 0.92 0.92

Pt2 PR Ps2 Tt2 TR Ts2 TwetBulb2 RH Tt2avg PR2avg
TIP 224.74 1.41 184.85 1241.86 1.11 1176.15 715.31 0.00
MEAN 218.28 1.37 178.89 1231.05 1.10 1164.73 712.77 0.00 1233.37 1.38
HUB 216.02 1.36 174.29 1227.19 1.10 1156.05 711.87 0.00

Alfa2 Beta FLO Beta BLADE Deviat Slip F. DiffFct Solidity Convergence Iter
TIP 43.55 44.70 40.30 4.40 0.93 0.52 1.50 0.00006 33
MEAN 42.42 40.17 35.60 4.57 0.93 0.49 1.73 0.00010 27
HUB 43.43 32.27 27.30 4.97 0.93 0.48 2.00 0.00058 22

Station 3. STATOR ROW #1
blockage3 XBladeGap Vane# Area3 Tt3 TwetBulb3 MeltFract Area3UB
0.9500 0.1520 74 15.0130 1233.3678 713.2232 0.0000 17.2921

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE
R3m C3 Cu3 Cm3 Ao3 Mach3 cp 2-3 Stat Ax Chd
4.1170 926.9187 610.5531 697.4261 1661.8674 0.5578 -0.0234 0.4666

Pt3 Ps3 Ts3 RH3 FloAlpha3 VaneAlpha3 Incid3
219.2762 177.9482 1164.1696 0.0000 41.2001 35.5000 5.7001

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Station 4. STATOR / VANED DIFFUSER  
 RCG Cth Cuth Cmth Aoth Machth cp 2-Th Stat Chord  
 4.1170 794.0248 194.7802 769.7637 1674.7329 0.4741 0.2073 0.4899

BlockageTh PtTh PsTh TsTh TwetBulbTh AreaTh w2-Th DiffFact4  
 0.9500 218.5336 187.7069 1182.6305 713.0366 13.7889 0.0284 0.2495

Station 5. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):  
 R4 C4 Cu4 Cm4 Ao4 Mach4 cp 3-4 Stator Gap  
 4.1017 931.2610 0.0000 931.2610 1660.6831 0.5608 -0.0682 0.1167

Blockage4 Ps4 Ts4 TwetBulb4 VaneAlpha4 Vane Thk4 w2-40D cp 2-4  
 0.9500 175.1279 1163.7079 712.4553 0.0000 0.0300 0.0854 -0.1045

STAGE EXIT CONDITIONS, STAGE 1  
 Eff4 Pt4 PR TR Tt4 Del T Ns Ns nondim  
 0.8661 216.2335 1.3578 1.1018 1233.3678 113.9778 162.0266 1.2560

Del Enthalpy Del\_H/U^2 GHP Reynolds# Specif Humid Area4 Area4UB  
 736437.125 0.459 1248.502 710722.0 0.00000 11.41982 16.49978

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 2  
 BLEED = 0.000 DPInc = 6.323 EfDer = 1.000 SH = 0.000000E+00

W act RPM act Pt Tt POTS POTH AeroBl  
 30.0000 32650.0000 216.2335 1233.3678 1.1000 0.9000 0.9800

W Kg/sec Wdry H2Ovap\_Lb/s H2Ovap\_g/m^3 m^3/min IWAR IWC  
 13.6364 30.0000 0.0000 0.0000 115.1641 0.0000 0.0000

W cor RPM cor GAMMA Cp R Blades THK ArealUB  
 3.1449 21173.6152 1.3793 0.2596 53.3483 60 0.0300 15.8491

CFM SCFM Al/A\* Areal A\* AthrRotor ChokeMargin AnnularArea  
 4066.981 23539.482 1.7058 15.5321 9.1055 8.7135 0.9569 15.6410

Station 6. ROTOR LEADING EDGE CONDITIONS, STAGE 2

R1 Stator Alfa Cl CU1 Cm1 Mabs Mrel Ulcor FlowCoeff  
 TIP 4.38 0.00 -0.02 683.09 -0.24 683.09 0.41 0.85 809.12 0.498  
 MEAN 4.08 0.00 -0.02 621.00 -0.21 621.00 0.37 0.78  
 HUB 3.77 0.00 -0.02 558.90 -0.19 558.90 0.33 0.72

BetaFlo BetaBlade Incid U1 W1 Ps1 Ts1 TwetBulb1 RH  
 TIP 61.30 54.70 6.60 1247.71 1422.67 193.34 1195.96 712.46 0.00  
 MEAN 61.92 55.60 6.32 1163.92 1319.41 197.18 1202.56 712.46 0.00  
 HUB 62.50 56.10 6.40 1073.60 1210.53 200.70 1208.31 712.46 0.00

Station 7. ROTOR EXIT CONDITIONS, STAGE 2 SOLUTION IS CONVERGED

B2 axial THK AeroBl Blades2 Area2 Area2UB  
 0.1000 0.0300 0.9500 60 13.3432 15.1118

R2 C2 Cu2 Cm2 Ao2 Mach2 Chord AxChord Rcircle  
 TIP 4.35 868.23 600.43 627.14 1745.98 0.50 0.66 0.50 2.81  
 MEAN 4.07 871.87 594.61 637.65 1739.89 0.50 0.71 0.52 2.20  
 HUB 3.77 890.03 610.93 647.24 1734.60 0.51 0.76 0.55 1.73

U2 W2 Wu2 MachRel2 DelRCu Eff2uC Eff2incC AvgREff Ws1/W2  
 TIP 1239.97 895.72 639.54 0.51 2614.04 0.92 0.91  
 MEAN 1159.77 852.06 565.16 0.49 2421.19 0.92 0.91 0.91 1.68  
 HUB 1073.60 795.60 462.67 0.46 2302.70 0.92 0.91

Pt2 PR Ps2 Tt2 TR Ts2 TwetBulb2 RH Tt2avg PR2avg  
 TIP 290.93 1.35 246.26 1348.00 1.09 1287.62 738.93 0.00  
 MEAN 284.86 1.32 240.52 1339.55 1.09 1278.65 737.00 0.00 1340.63 1.32  
 HUB 281.17 1.30 235.52 1334.35 1.08 1270.89 735.81 0.00

Alfa2 Beta FLO Beta BLADE Deviat Slip F. DiffFct Solidity Convergence Iter  
 TIP 43.75 45.56 41.20 4.36 0.93 0.52 1.45 0.00014 46  
 MEAN 43.00 41.55 37.00 4.55 0.93 0.49 1.66 0.00031 36  
 HUB 43.35 35.56 30.70 4.86 0.93 0.47 1.93 0.00029 35

Station 8. STATOR ROW #1

blockage3	XBladeGap	Vane#	Area3	Tt3	TwetBulb3	MeltFract	Area3UB
0.9500	0.1383	81	14.7994	1340.6338	737.1583	0.0000	17.2001

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE

R3m	C3	Cu3	Cm3	Ao3	Mach3	cp 2-3	Stat Ax Chd
4.1153	816.2090	588.1330	565.9478	1743.7570	0.4681	0.1230	0.4223

Pt3	Ps3	Ts3	RH3	FloAlpha3	VaneAlpha3	Incid3
285.2036	246.0433	1287.6243	0.0000	46.1013	38.5000	7.6013

Station 9. STATOR / VANED DIFFUSER

RCG	Cth	Cuth	Cmth	Aoth	Machth	cp 2-Th	Stat Chord
4.1153	691.8416	183.7228	667.0013	1753.2676	0.3946	0.3376	0.4473

BlockageTh	PtTh	PsTh	TsTh	TwetBulbTh	AreaTh	w2-Th	DiffFact4
0.9500	284.3996	255.9223	1302.6188	736.9893	12.8558	0.0279	0.1153

Station 10. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):

R4	C4	Cu4	Cm4	Ao4	Mach4	cp 3-4	Stator Gap
4.0528	951.3206	0.0000	951.3206	1729.8645	0.5499	-0.4221	0.1056

Blockage4	Ps4	Ts4	TwetBulb4	VaneAlpha4	Vane Thk4	w2-4OD	cp 2-4
0.9500	229.5142	1268.8771	736.2527	0.0000	0.0300	0.1054	-0.2507

STAGE EXIT CONDITIONS, STAGE 2

Eff4	Pt4	PR	TR	Tt4	Del T	Ns	Ns nondim
0.8523	280.9202	1.2992	1.0870	1340.6338	107.2660	148.8458	1.1538

Del Enthalpy	Del_H/U^2	GHP	Reynolds#	Specif Humid	Area4	Area4UB
702000.625	0.457	1190.121	648676.6	0.00000	9.30090	13.99603

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 3

BLEED = 0.000 DPInc = 6.220 EfDer = 1.000 SH = 0.000000E+00

W act	RPM act	Pt	Tt	POTS	POTH	AeroBl
30.0000	32650.0000	280.9202	1340.6338	1.1000	0.9000	0.9800

W Kg/sec	Wdry	H2Ovap_Lb/s	H2Ovap_g/m^3	m^3/min	IWAR	IWC
13.6364	30.0000	0.0000	0.0000	95.6428	0.0000	0.0000

W cor	RPM cor	GAMMA	Cp	R	Blades	THK	ArealUB
2.4043	21317.7715	1.3732	0.2628	53.3483	68	0.0300	13.3698

CFM	SCFM	A1/A*	Areal	A*	AthrRotor	ChokeMargin	AnnularArea
3377.593	23539.482	1.7953	13.1024	7.2984	7.1706	0.9825	13.1282

Station 11. ROTOR LEADING EDGE CONDITIONS, STAGE 3

TIP	R1	Stator	Alfa	C1	CU1	Cm1	Mabs	Mrel	U1cor	FlowCoeff
4.29	0.00	-0.02	672.11	-0.23	672.11	0.38	0.79	797.46	0.500	
MEAN	4.04	0.00	-0.02	611.01	-0.21	611.01	0.35	0.74		
HUB	3.77	0.00	-0.02	549.91	-0.19	549.91	0.31	0.68		

TIP	BetaFlo	BetaBlade	Incid	U1	W1	Ps1	Ts1	TwetBulb1	RH
61.18	54.90	6.28	1221.42	1394.33	254.32	1304.85	736.25	0.00	
MEAN	62.02	55.80	6.22	1149.88	1302.33	258.79	1310.79	736.25	0.00
HUB	62.88	56.40	6.48	1073.60	1206.41	262.90	1316.66	736.25	0.00

Station 12. ROTOR EXIT CONDITIONS, STAGE 3 SOLUTION IS CONVERGED

B2 axial	THK	AeroBl	Blades2	Area2	Area2UB
0.1000	0.0300	0.9500	68	11.1393	12.7570

TIP	R2	C2	Cu2	Cm2	Ao2	Mach2	Chord	AxChord	Rcircle
4.26	866.89	587.27	637.66	1809.77	0.48	0.57	0.43	2.23	
MEAN	4.02	871.21	584.95	645.64	1805.13	0.48	0.62	0.45	1.84
HUB	3.77	891.67	607.91	652.31	1801.48	0.49	0.66	0.48	1.49

TIP	U2	W2	Wu2	MachRel2	DelRCu	Eff2uC	Eff2incC	AvgREff	Wsl/W2
1214.81	894.65	627.53	0.49	2504.90	0.92	0.91			
MEAN	1146.38	855.60	561.43	0.47	2354.35	0.92	0.91	0.91	1.64
HUB	1073.60	801.48	465.68	0.44	2291.33	0.92	0.91		

	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
TIP	365.33	1.30	313.11	1449.11	1.08	1389.62	760.89	0.00		
MEAN	359.79	1.28	307.65	1442.59	1.08	1382.51	759.42	0.00	1443.86	1.28
HUB	357.49	1.27	303.28	1439.87	1.07	1376.92	758.80	0.00		

	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter
TIP	42.64	44.54	40.20	4.34	0.93	0.50	1.45	0.00065	39
MEAN	42.18	41.01	36.50	4.51	0.93	0.48	1.66	0.00000	36
HUB	42.98	35.52	30.70	4.82	0.93	0.47	1.91	0.00022	34

Station 13. STATOR ROW #1

blockage3	XBladeGap	Vane#	Area3	Tt3	TwetBulb3	MeltFract	Area3UB
0.9500	0.1205	94	10.6454	1443.8579	759.6096	0.0000	12.6149

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE

R3m	C3	Cu3	Cm3	Ao3	Mach3	cp 2-3	Stat Ax Chd
4.0256	896.7473	584.6310	679.9722	1801.5952	0.4978	-0.0453	0.3578

Pt3	Ps3	Ts3	RH3	FloAlpha3	VaneAlpha3	Incid3
360.3426	305.2574	1380.6364	0.0000	40.6885	30.5000	10.1885

Station 14. STATOR / VANED DIFFUSER

RCG	Cth	Cuth	Cmth	Aoth	Machth	cp 2-Th	Stat Chord
4.0256	750.0560	158.5054	733.1166	1813.8141	0.4135	0.2236	0.3708

BlockageTh	PtTh	PsTh	TsTh	TwetBulbTh	AreaTh	w2-Th	DiffFact4
0.9500	358.9107	319.8323	1399.6471	759.3522	10.0564	0.0371	0.0652

Station 15. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):

R4	C4	Cu4	Cm4	Ao4	Mach4	cp 3-4	Stator Gap
4.0107	1063.8884	0.0000	1063.8884	1783.9906	0.5964	-0.4732	0.0894

Blockage4	Ps4	Ts4	TwetBulb4	VaneAlpha4	Vane Thk4	w2-4OD	cp 2-4
0.9500	279.1935	1355.1078	758.3528	0.0000	0.0300	0.1415	-0.5452

STAGE EXIT CONDITIONS, STAGE 3

Eff4	Pt4	PR	TR	Tt4	Del T	Ns	Ns nondim
0.8322	353.3931	1.2580	1.0770	1443.8579	103.2241	138.2431	1.0717

Del Enthalpy	Del_H/U^2	GHP	Reynolds#	Specif Humid	Area4	Area4UB
684200.125	0.464	1159.943	718215.8	0.00000	7.30153	11.86386

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 4

BLEED = 0.000 DPInc = 5.909 EfDer = 0.999 SH = 0.000000E+00

W act	RPM act	Pt	Tt	POTS	POTH	AeroBl
30.0000	32650.0000	353.3931	1443.8579	1.1000	0.9000	0.9800

W Kg/sec	Wdry	H2Ovap_Lb/s	H2Ovap_g/m^3	m^3/min	IWAR	IWC
13.6364	30.0000	0.0000	0.0000	81.6310	0.0000	0.0000

W cor	RPM cor	GAMMA	Cp	R	Blades	THK	ArealUB
1.9025	21416.2500	1.3672	0.2662	53.3483	80	0.0300	11.2780

CFM	SCFM	A1/A*	Areal	A*	AthrRotor	ChokeMargin	AnnularArea
2882.772	23539.482	1.8383	11.0524	6.0123	5.9132	0.9835	10.9962

Station 16. ROTOR LEADING EDGE CONDITIONS, STAGE 4

TIP	R1	Stator	Alfa	C1	CU1	Cm1	Mabs	Mrel	U1cor	FlowCoeff
4.21	0.00	-0.02	679.44	-0.23	679.44	0.37	0.76	786.21	0.515	
MEAN	3.99	0.00	-0.02	617.67	-0.21	617.67	0.34	0.71		
HUB	3.77	0.00	-0.02	555.90	-0.19	555.90	0.30	0.66		

	BetaFlo	BetaBlade	Incid	U1	W1	Ps1	Ts1	TwetBulb1	RH
TIP	60.46	54.70	5.76	1198.65	1378.03	321.58	1407.72	758.35	0.00
MEAN	61.51	55.60	5.91	1137.84	1294.87	326.94	1413.35	758.35	0.00
HUB	62.63	56.50	6.13	1073.60	1209.15	331.86	1419.65	758.35	0.00

Station 17. ROTOR EXIT CONDITIONS, STAGE 4 SOLUTION IS CONVERGED

B2 axial	THK	AeroBl	Blades2	Area2	Area2UB
0.1000	0.0300	0.9500	80	9.3027	10.8322



	R2	C2	Cu2	Cm2	Ao2	Mach2	Chord	AxChord	Rcircle
TIP	4.19	877.91	575.34	663.11	1867.81	0.47	0.48	0.37	1.71
MEAN	3.98	887.92	584.18	668.67	1864.71	0.48	0.52	0.38	1.45
HUB	3.77	907.43	607.75	673.84	1861.97	0.49	0.56	0.41	1.22

	U2	W2	Wu2	MachRel2	DelRCu	Eff2uC	Eff2incC	AvgREff	Wsl/W2
TIP	1193.72	906.70	618.38	0.49	2411.44	0.92	0.91		
MEAN	1135.25	866.49	551.07	0.46	2328.46	0.92	0.91	0.91	1.59
HUB	1073.60	819.19	465.85	0.44	2290.72	0.92	0.91		

	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
TIP	447.26	1.27	385.72	1546.99	1.07	1486.69	781.49	0.00		
MEAN	443.76	1.26	381.24	1543.44	1.07	1481.76	780.70	0.00	1544.08	1.26
HUB	442.17	1.25	377.20	1541.83	1.07	1477.41	780.34	0.00		

	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter
TIP	40.95	43.00	38.70	4.30	0.93	0.49	1.45	0.00063	35
MEAN	41.14	39.49	35.00	4.49	0.93	0.47	1.65	0.00099	36
HUB	42.05	34.66	29.90	4.76	0.93	0.46	1.90	0.00015	30

Station 18. STATOR ROW #1

blockage3	XBladeGap	Vane#	Area3	Tt3	TwetBulb3	MeltFract	Area3UB
0.9500	0.1023	104	9.0414	1544.0840	780.7478	0.0000	10.8720

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE

R3m	C3	Cu3	Cm3	Ao3	Mach3	cp 2-3	Stat Ax Chd
3.9910	904.5092	583.2123	691.3756	1861.6190	0.4859	-0.0346	0.3188

Pt3	Ps3	Ts3	RH3	FloAlpha3	VaneAlpha3	Incid3
443.7663	379.0558	1480.5123	0.0000	40.1494	30.0000	10.1494

Station 19. STATOR / VANED DIFFUSER

RCG	Cth	Cuth	Cmth	Aoth	Machth	cp 2-Th	Stat Chord
3.9910	776.3097	161.4039	759.3455	1872.0052	0.4147	0.1988	0.3300

BlockageTh	PtTh	PsTh	TsTh	TwetBulbTh	AreaTh	w2-Th	DiffFact4
0.9500	442.1228	393.9142	1497.2725	780.4911	8.4329	0.0361	0.0551

Station 20. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):

R4	C4	Cu4	Cm4	Ao4	Mach4	cp 3-4	Stator Gap
3.9887	1079.1113	0.0000	1079.1113	1843.9775	0.5852	-0.4972	0.0797

Blockage4	Ps4	Ts4	TwetBulb4	VaneAlpha4	Vane Thk4	w2-40D	cp 2-4
0.9500	346.8786	1453.8099	779.3694	0.0000	0.0300	0.1491	-0.5476

STAGE EXIT CONDITIONS, STAGE 4

Eff4	Pt4	PR	TR	Tt4	Del T	Ns	Ns nondim
0.8233	435.0002	1.2309	1.0694	1544.0840	100.2261	129.3042	1.0024

Del Enthalpy	Del_H/U^2	GHP	Reynolds#	Specif Humid	Area4	Area4UB
672614.875	0.472	1140.302	782384.1	0.00000	6.21592	10.75588

OVERALL EXIT CONDITIONS; ALL 4 STAGES

Del Enthal	DelHT/U1^2	GHP	MassFloSlcor	OPR	Efficiency	RotorlInc	TR	AxHubLen
2795253.00	1.7192	4738.8682	4.068	2.7316	0.82821	6.0148	1.3794	4.522

### 7.3.2 Large Civil Transport Rotor Six-Stage Low-Pressure Compressor and Four-Stage Axial High-Pressure Compressor

Figure 24 illustrates the six-stage LPC and four-stage HPC configuration for the Large Civil Transport Rotor Engine.

### 7.4 Example Case (4): Large Civil Transport Rotor Engine

The input and output files in this section are those used for the analysis of option 2 for the HPC consisting of single-stage centrifugal compressor for this example case.

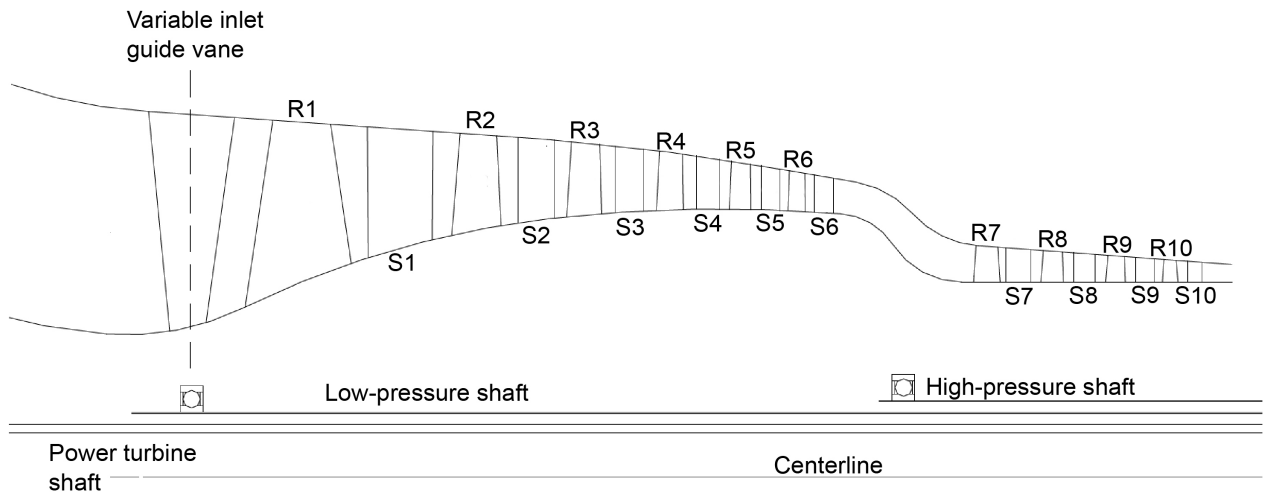


Figure 24.—Flow path for the six-stage axial low-pressure compressor with the four-stage axial high-pressure compressor of the Large Civil Transport Rotor Engine, option 1.

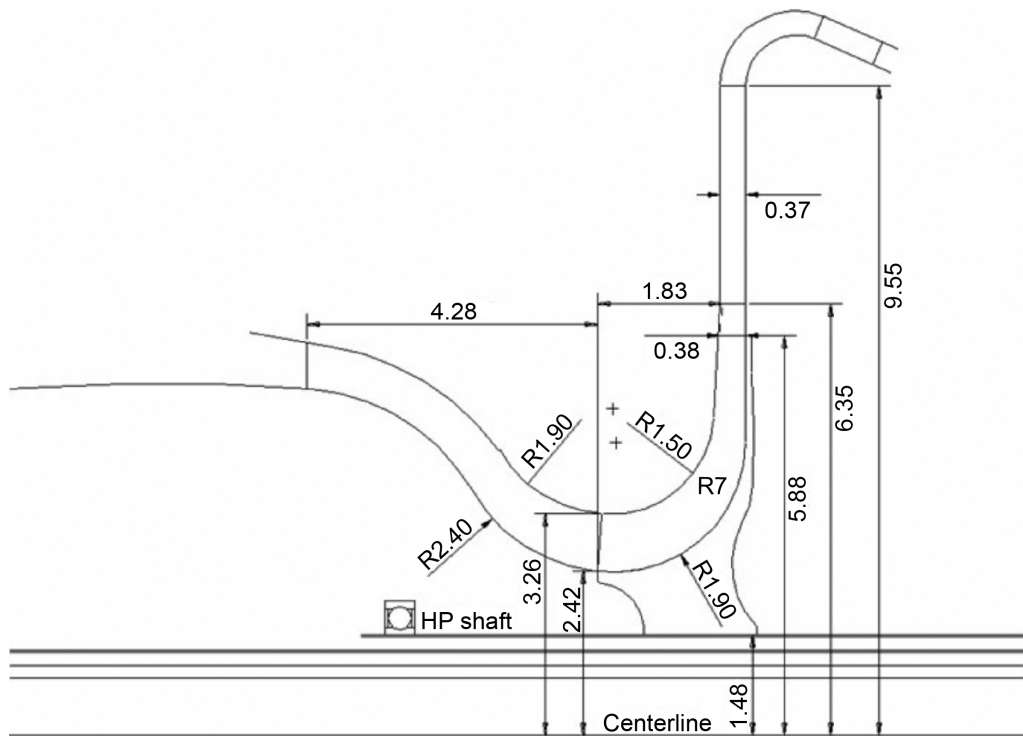


Figure 25.—Flow path for single-stage centrifugal high-pressure (HP) compressor of the Large Civil Transport Rotor Engine, option 2 (Ref. 12).

#### 7.4.1 High-Pressure Compressor: Option 2 One Centrifugal Stage

Figure 25 illustrates the one-stage centrifugal HPC configuration for the Large Civil Transport Rotor Engine. The centrifugal impeller inlet and exit velocity diagrams are illustrated in Figure 26.

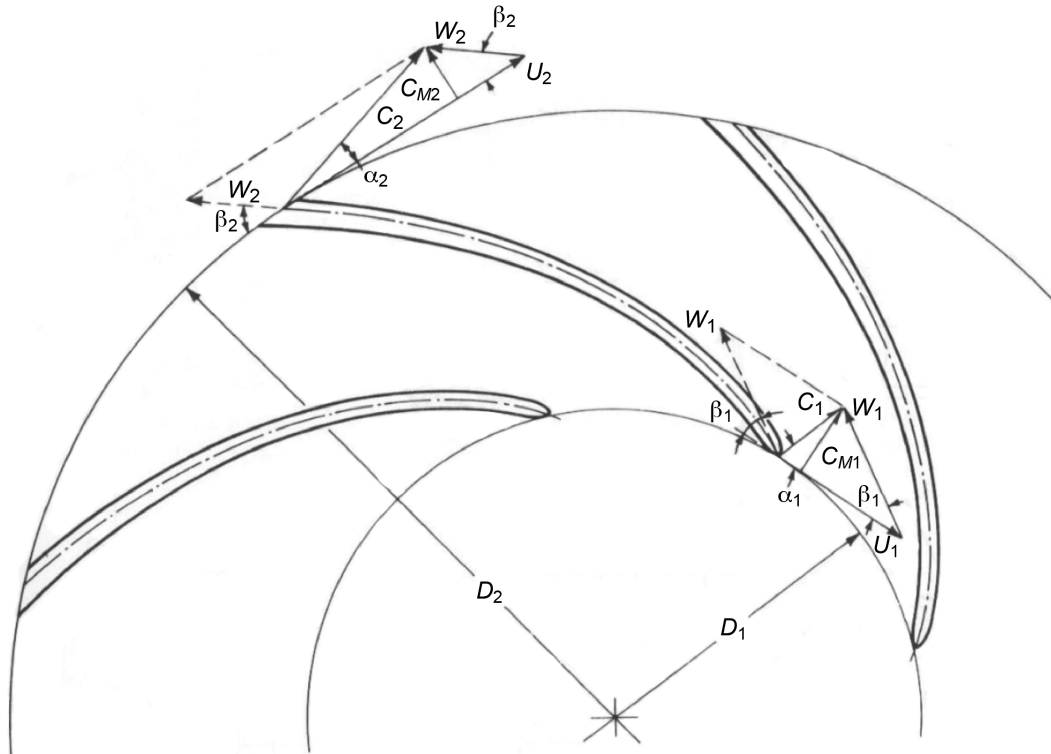


Figure 26.—Centrifugal impeller inlet and exit velocity diagrams. The impeller exit velocity triangles are the same as for an axial rotor, except the definition of meridional changes from the axial direction to the radial direction. Absolute velocity is  $C$ ; rotor peripheral velocity is  $U$ ; relative velocity is  $W$ ; absolute flow angle is  $\alpha$ ; relative flow angle is  $\beta$ . Subscripts: leading edge is 1; trailing edge is 2; meridional is  $M$ .

### 7.4.1.1 Input File

comdes.dat

```

Single Centrifugal SRW HPC Option # 2 Low Ns 1-20-2010
30.0 159.25 1119.39 38620.0 Wact Pt1 Tt1 RPM
5 3 0 0.00 Fluid Stages IDUMMY DUMMY
0.00 46.900 000.00 0.00 DropDia InDist RHinlet IWC ***icing***
2.70 0.00 406.44 0.00 Psin DUMMY Tsin DUMMY ***icing***
607.8 0.00 00000.00 0 Vin DUMMY Altitude IDUMMY ***icing***
3.40 2.41 5.88 5.88 1 Rltip Rlhub R2tip R2hub
0.3300 0.380 30 0.0300 1.0 Blaxial B2axial NBLAD THK CFS
0.0 0.0000 0.000 1.0000 1.0 StatorT StatorM StatorH FSGV CFMid
0.9800 0.9500 1.1000 0.9000 1.0 AeroB1 AeroB2 POTS POTH CFHub
0.842 0.9220 0.917 0.0000 EfficiT EfficiM EfficiH BLEED
0.00 0.00 0.00 0.00 DUMMY DUMMY DUMMY DUMMY
0.8500 0.16 1.460 0.00 SlipF DiffLoss Solidity RPM2
51.50 50.60 48.00 0.0 BetaBlade1T BetaBlade1M BetaBlade1H expo3
23.50 23.50 23.50 2 BetaBlade2T BetaBlade2M BetaBlade2H stat2
6.35 6.35 0.38 67.5 R3s R3h B3X BetB3
74 0.03 0.95 0.01 Blan3 Blath3 Blok3 Wbar23
6.60 10.60 0.95 0.015 RCG Athrt BlokTH Wbar2T
9.55 9.55 0.37 55.0 R4S R4H B4X BetB4
74 0.03 0.95 1.30 Blan4 Blath4 Blok4 SolidStat
10.50 10.10 0.00 50.0 R5s R5h B5X BetB5
95 0.03 0.95 0.01 Blan5 Blath5 Blok5 Wbar56
0.00 0.0000 0.95 0.015 RCG2 Athrt2 BlokTH Wbar5T
10.50 10.10 0.0 0.0 R6S R6H B6X BetB6
95 0.03 0.95 1.30 Blan4 Blath4 Blok4 SolidStat
10.50 10.10 0.00 1.0 R7S R7h B7X Length67
2 0.01 0.98 0.0000 Blan7 Blath7 Blok7 Wbar67
10.50 10.10 0.0 1.0 R8S R8h B8X Length78
2 0.01 0.98 0.00 Blan8 Blath8 Blok8 Wbar78

```

## 7.4.1.2 Output File

comdes.out

```

*****
***** AXIAL & CENTRIFUGAL COMPRESSOR BLADE DESIGN 1-D ****
***** COMDES-MELT Release Version 1 *****
***** Gasplus Fluid Properties *****
*****

Single Centrifugal SRW HPC Option # 2 Low Ns 1-20-2010

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 1
BLEED = 0.000 DPInc = 6.045 EfDer = 0.999 SH = 0.000000E+00

W act RPM act Pt Tt POTS POTH AeroBl
30.0000 38620.0000 159.2484 1119.3900 1.1000 0.9000 0.9800

W Kg/sec Wdry H2Ovap_Lb/s H2Ovap_g/m^3 m^3/min IWAR IWC
13.6364 30.0000 0.0000 0.0000 144.0139 0.0000 0.0000

W cor RPM cor GAMMA Cp R Blades THK ArealUB
4.0681 26289.3398 1.3854 0.2564 53.3483 30 0.0300 19.0476

CFM SCFM Al/A* Areal A* AthrRotor ChokeMargin AnnularArea
5085.802 23539.482 1.5693 18.6666 11.8946 12.3309 1.0367 18.0701

Station 1. ROTOR LEADING EDGE CONDITIONS, STAGE 1

R1 Stator Alfa C1 CU1 Cm1 Mabs Mrel Ulcor FlowCoeff
TIP 3.40 0.00 -0.02 719.28 -0.25 719.28 0.45 0.85 780.00 0.571
MEAN 2.95 0.00 -0.02 653.89 -0.23 653.89 0.41 0.74
HUB 2.41 0.00 -0.02 588.50 -0.20 588.50 0.37 0.62

BetaFlo BetaBlade Incid U1 W1 Ps1 Ts1 TwetBulb1 RH
TIP 57.89 51.50 6.39 1145.88 1353.13 138.83 1077.44 686.19 0.00
MEAN 56.65 50.60 6.05 993.16 1189.28 142.23 1084.72 686.19 0.00
HUB 54.08 48.00 6.08 812.23 1003.18 145.36 1091.29 686.19 0.00

Station 2. ROTOR EXIT CONDITIONS, STAGE 1 SOLUTION IS CONVERGED
B2 axial THK AeroBl Blades2 Area2 Area2UB
0.3800 0.0300 0.9500 30 13.0123 14.0392

R2 C2 Cu2 Cm2 Ao2 Mach2 Chord AxChord Rcircle
TIP 5.88 1544.75 1419.36 609.65 1801.20 0.86 1.80 1.60 3.72
MEAN 5.88 1546.23 1443.67 553.75 1805.90 0.86 1.99 1.67 4.24
HUB 5.88 1546.07 1442.23 557.05 1805.62 0.86 2.20 1.78 5.18

U2 W2 Wu2 MachRel2 DelRCu Eff2uC Eff2incC AvgREff Wsl/W2
TIP 1981.70 829.40 562.34 0.46 8346.67 0.84 0.84
MEAN 1981.70 772.08 538.03 0.43 8489.44 0.92 0.92 0.89 1.71
HUB 1981.70 775.46 539.47 0.43 8480.80 0.92 0.92

Pt2 PR Ps2 Tt2 TR Ts2 TwetBulb2 RH Tt2avg PR2avg
TIP 443.25 2.78 275.24 1557.69 1.39 1364.30 781.61 0.00
MEAN 489.83 3.08 304.61 1565.18 1.40 1371.43 789.14 0.00 1562.53 2.97
HUB 486.80 3.06 302.71 1564.73 1.40 1371.01 788.67 0.00

Alfa2 Beta FLO Beta BLADE Deviat Slip F. DiffFct Solidity Convergence Iter
TIP 66.76 42.69 23.50 19.19 0.85 0.84 1.46 0.00003 162
MEAN 69.01 44.18 23.50 20.68 0.85 0.60 3.22 0.00033 199
HUB 68.88 44.08 23.50 20.58 0.85 0.46 4.35 0.00018 205

Station 3. STATOR ROW #1
blockage3 XBladeGap Vane# Area3 Tt3 TwetBulb3 MeltFract Area3UB
0.9500 0.4458 74 13.6018 1562.5331 786.2665 0.0000 15.1613

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE
R3m C3 Cu3 Cm3 Ao3 Mach3 cp 2-3 Stat Ax Chd
6.3500 1435.9899 1336.8154 524.3961 1814.6038 0.7914 0.0535 0.8182

Pt3 Ps3 Ts3 RH3 FloAlpha3 VaneAlpha3 Incid3
471.5013 314.2023 1400.3114 0.0000 68.5813 67.5000 1.0813

```

Station	4. STATOR / VANED DIFFUSER						
RCG	Cth	Cuth	Cmth	Aoth	Machth	cp 2-Th	Stat Chord
6.6000	1416.8678	1227.0436	708.4338	1816.6155	0.7799	0.1175	0.7009
BlockageTh	PtTh	PsTh	TsTh	TwetBulbTh	AreaTh	w2-Th	DiffFact4
0.9500	467.6732	315.2260	1405.1252	785.6934	10.6000	0.0314	0.7621
Station	5. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):						
R4	C4	Cu4	Cm4	Ao4	Mach4	cp 3-4	Stator Gap
9.5500	545.7169	447.0251	313.0103	1901.1146	0.2871	0.4658	0.0843
Blockage4	Ps4	Ts4	TwetBulb4	VaneAlpha4	Vane Thk4	w2-4OD	cp 2-4
0.9500	387.4723	1539.1947	776.5165	55.0000	0.0300	0.3547	0.5208
Pt4	Tt4	Area4					
409.7647	1562.5331	20.3112					
Station	6. STATOR ROW #2						
blockage5	XStatGap	Vane#	Area5	Tt5	TwetBulb5	MeltFract	
0.9500	0.2046	95	21.1900	1562.5331	776.5165	0.0000	
ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE							
R5m	C5	Cu5	Cm5	Ao5	Mach5	cp4-5	StatAxChd
10.3019	520.3502	414.3966	314.7058	1897.3145	0.2743	0.0892	0.8182
Pt5	Ps5	Ts5	RH5	FloAlpha5	VaneAlpha5	Incid5	
409.7647	389.4612	1541.6371	0.0000	52.7858	50.0000	2.7858	
Station	7. STATOR / VANED #2 DIFFUSER THROAT IS NOT CHOKED:						
RCG	Cth	Cuth	Cmth	Aoth2	Machth	cp4-Th	StatChord
10.3019	456.8296	228.4148	395.6260	1900.1666	0.2404	0.2839	0.9028
BlockageTh	PtTh	PsTh	TsTh	TwetBulbTh	AreaTh	w2-Th	TtTh
0.9500	409.5075	393.8015	1546.4319	776.4734	15.8859	0.0115	1562.5331
Station	8. VANED DIFFUSER #2 EXIT:						
R6	C6	Cu6	Cm6	Ao6	Mach6	cp5-6	StatorGap
10.3019	291.5694	0.0000	291.5694	1905.8323	0.1530	0.6699	0.0843
Blockage6	Ps6	Ts6	TwetBulb6	MeltFract	Vane Thk6	w4-6OD	cp 2-6
0.9500	403.0620	1555.9778	776.4747	0.0000	0.0300	0.0112	0.6079
STAGE EXIT CONDITIONS, STAGE 1							
Eff6	Pt6	PR6	TR	Tt6	Del T	Ns	Ns nondim
0.7166	409.5146	2.5715	1.3959	1562.5331	443.1431	67.7343	0.5251
Del Enthalpy	Del_H/U^2	GHP	Reynolds#	Specif Humid	DF6	W2-6	Area6
2935275.000	0.7474	4976.251	960275.06	0.00000	0.7460	0.3561	21.19

## 7.4.2 Large Civil Transport Rotor Six-Stage Low-Pressure Compressor and One Centrifugal High-Pressure Compressor Stage

Figure 27 illustrates the six-stage LPC and single-stage centrifugal HPC configuration for the Large Civil Transport Rotor Engine.

## 7.5 Example Case (5): Helium Compressor

The input and output files in this section are those used for the analysis of a five-stage helium compressor. The flow path for the compressor is illustrated in Figure 28.

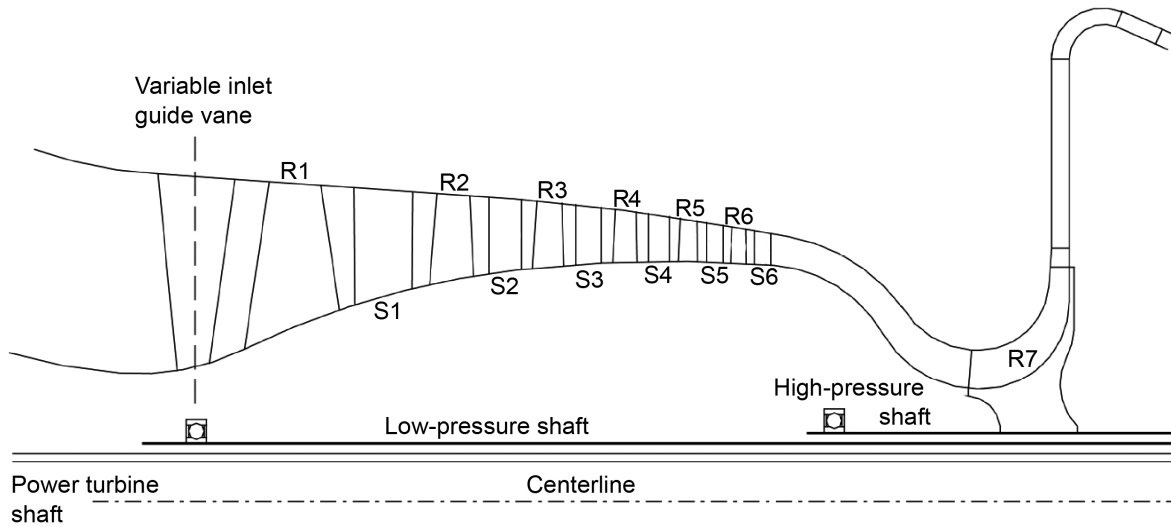


Figure 27.—Flow path for the six-stage axial low-pressure compressor with the single-stage centrifugal high-pressure compressor of the Large Civil Transport Rotor Engine, option 2.

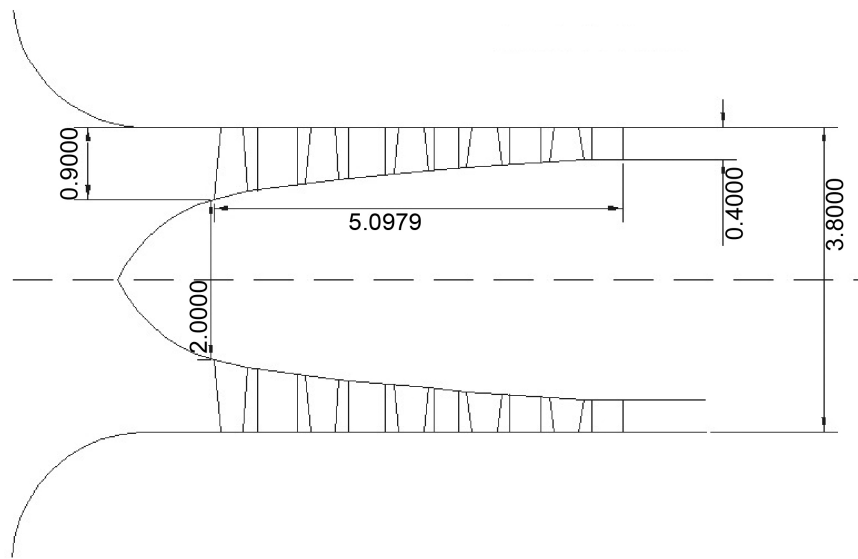


Figure 28.—Flow path for the five-stage axial helium compressor. All dimensions in inches. Shaft speed (actual) = 68,000 RPM; shaft speed (corrected) = 163,247 RPM; pressure ratio = 3.47; temperature ratio = 1.8733; efficiency = 81.2; horsepower = 14,185 HP; mass flow (actual) = 97.5 lbm/s; mass flow (corrected) = 0.7036 lbm/s; tip speed (actual) = 1,127 ft/s; tip speed (corrected stage 1) = 2,706 ft/s; inlet pressure = 848.5 psia; inlet temperature = 90 R.

### 7.5.1 Input File

#### comdes.dat

Helium Axial Compressor 7-7-2014

97.5	848.47	90.0	68000.	Wact	Pt1	Tt1	RPM		
6	5	0	0.00	Fluid	Stages	IDUMMY	DUMMY		
0.00	5.920	000.00	0.00	DropDia	InDist	RHinlet	IWC	***icing***	
780.0	0.00	88.0	0.00	Psin	DUMMY	Tsin	DUMMY	***icing***	
472.0	0.00	000.00	0	Vin	DUMMY	Altitude	IDUMMY	***icing***	
1.90	1.0	1.9	1.1	1 Rltip	Rlhub	R2tip	R2hub		
0.05	0.05	40	0.020	1.0	Blaxial	B2axial	NBLAD	THK	CFS
0.0	0.0000	0.000	1.0000	0.9	StatorT	StatorM	StatorH	FSGV	CFMid

0.9500	0.9500	1.0000	1.0000	0.9	AeroB1	AeroB2	POTS	POTH	CFHub
0.880	0.875	0.850	0.0000	EfficiT	EfficiM	EfficiH	BLEED		
0.00	0.00	0.00	0.00	DUMMY	DUMMY	DUMMY	DUMMY		
0.915	0.03	1.8	0.00	SlipF	DiffLoss	Solidity	RPM2		
57.5	52.0	40.5	2.0	BetaBlade1T	BetaBlade1M	BetaBlade1H	expo3		
46.5	31.5	3.5	0	BetaBlade2T	BetaBlade2M	BetaBlade2H	stat2		
1.9	1.1161	0.05	38.5	R3s	R3h	B3X	BetB3		
31	0.02	0.95	0.00	Blan3	Blath3	Blok3	Wbar23		
1.9	0.0	0.97	0.015	RCG	Athrt	BlokTH	Wbar2T		
1.9	1.03	0.05	0.0	R4S	R4H	B4X	BetB4		
31	0.01	0.98	1.36	Blan4	Blath4	Blok4	SolidStat1		
1.90	1.195	1.9	1.25	2 R1tip	Rlhub	R2tip	R2hub		
0.05	0.05	40	0.020	1.0	Blaxial	B2axial	NBLAD	THK	CFS
0.0	0.0000	0.000	1.0000	0.9	StatorT	StatorM	StatorH	FSGV	CFMid
0.9500	0.9500	1.0000	1.0000	0.9	AeroB1	AeroB2	POTS	POTH	CFHub
0.880	0.875	0.850	0.0000	EfficiT	EfficiM	EfficiH	BLEED		
0.00	0.00	0.00	0.00	DUMMY	DUMMY	DUMMY	DUMMY		
0.915	0.03	1.8	0.00	SlipF	DiffLoss	Solidity	RPM2		
57.0	52.5	44.5	2.0	BetaBlade1T	BetaBlade1M	BetaBlade1H	expo3		
44.5	31.5	7.5	0	BetaBlade2T	BetaBlade2M	BetaBlade2H	stat2		
1.9	1.2585	0.05	40.5	R3s	R3h	B3X	BetB3		
33	0.02	1.00	0.00	Blan3	Blath3	Blok3	Wbar23		
0.0	0.0	0.97	0.015	RCG	Athrt	BlokTH	Wbar2T		
1.9	1.25	0.05	0.0	R4S	R4H	B4X	BetB4		
33	0.01	0.98	1.36	Blan4	Blath4	Blok4	SolidStat1		
1.90	1.3107	1.9	1.35	3 R1tip	Rlhub	R2tip	R2hub		
0.05	0.05	40	0.020	1.0	Blaxial	B2axial	NBLAD	THK	CFS
0.0	0.0000	0.000	1.0000	0.9	StatorT	StatorM	StatorH	FSGV	CFMid
0.9500	0.9500	1.0000	1.0000	0.9	AeroB1	AeroB2	POTS	POTH	CFHub
0.880	0.875	0.850	0.0000	EfficiT	EfficiM	EfficiH	BLEED		
0.00	0.00	0.00	0.00	DUMMY	DUMMY	DUMMY	DUMMY		
0.915	0.03	1.8	0.00	SlipF	DiffLoss	Solidity	RPM2		
56.5	53.0	47.0	2.0	BetaBlade1T	BetaBlade1M	BetaBlade1H	expo3		
43.5	31.5	7.5	0	BetaBlade2T	BetaBlade2M	BetaBlade2H	stat2		
1.9	1.3578	0.05	40.5	R3s	R3h	B3X	BetB3		
51	0.02	1.00	0.00	Blan3	Blath3	Blok3	Wbar23		
0.0	0.0	0.97	0.015	RCG	Athrt	BlokTH	Wbar2T		
1.9	1.3889	0.05	0.0	R4S	R4H	B4X	BetB4		
51	0.01	0.98	1.36	Blan4	Blath4	Blok4	SolidStat1		
1.90	1.40	1.9	1.4302	4 R1tip	Rlhub	R2tip	R2hub		
0.05	0.05	40	0.020	1.0	Blaxial	B2axial	NBLAD	THK	CFS
0.0	0.0000	0.000	1.0000	0.9	StatorT	StatorM	StatorH	FSGV	CFMid
0.9500	0.9500	1.0000	1.0000	0.9	AeroB1	AeroB2	POTS	POTH	CFHub
0.880	0.875	0.850	0.0000	EfficiT	EfficiM	EfficiH	BLEED		
0.00	0.00	0.00	0.00	DUMMY	DUMMY	DUMMY	DUMMY		
0.915	0.03	1.8	0.00	SlipF	DiffLoss	Solidity	RPM2		
56.5	53.0	48.0	2.0	BetaBlade1T	BetaBlade1M	BetaBlade1H	expo3		
40.5	29.5	7.5	0	BetaBlade2T	BetaBlade2M	BetaBlade2H	stat2		
1.9	1.4365	0.05	42.0	R3s	R3h	B3X	BetB3		
40	0.02	0.95	0.00	Blan3	Blath3	Blok3	Wbar23		
0.0	0.0	0.97	0.015	RCG	Athrt	BlokTH	Wbar2T		
1.9	1.4624	0.05	0.0	R4S	R4H	B4X	BetB4		
40	0.01	0.98	1.36	Blan4	Blath4	Blok4	SolidStat1		
1.90	1.4701	1.9	1.50	5 R1tip	Rlhub	R2tip	R2hub		
0.05	0.05	40	0.020	1.0	Blaxial	B2axial	NBLAD	THK	CFS
0.0	0.0000	0.000	1.0000	0.9	StatorT	StatorM	StatorH	FSGV	CFMid
0.9500	0.9500	1.0000	1.0000	0.9	AeroB1	AeroB2	POTS	POTH	CFHub
0.880	0.875	0.850	0.0000	EfficiT	EfficiM	EfficiH	BLEED		
0.00	0.00	0.00	0.00	DUMMY	DUMMY	DUMMY	DUMMY		
0.915	0.03	1.8	0.00	SlipF	DiffLoss	Solidity	RPM2		
56.5	54.0	50.0	2.0	BetaBlade1T	BetaBlade1M	BetaBlade1H	expo3		
40.5	29.5	7.5	0	BetaBlade2T	BetaBlade2M	BetaBlade2H	stat2		
1.9	1.50	0.05	42.5	R3s	R3h	B3X	BetB3		
40	0.02	0.95	0.00	Blan3	Blath3	Blok3	Wbar23		
0.0	0.0	0.97	0.015	RCG	Athrt	BlokTH	Wbar2T		
1.9	1.50	0.05	0.0	R4S	R4H	B4X	BetB4		
40	0.01	0.98	1.36	Blan4	Blath4	Blok4	SolidStat1		
1.9	1.50	0.05	1.5	R7S	R7h	B7X	Length67		
11	0.01	0.98	0.020	Blan7	Blath7	Blok7	Wbar67		
1.9	1.50	0.05	1.1	R8S	R8h	B8X	Length78		
11	0.01	0.98	0.02	Blan8	Blath8	Blok8	Wbar78		

## 7.5.2 Output File

comdes.out

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*****
***** AXIAL & CENTRIFUGAL COMPRESSOR BLADE DESIGN 1-D *****
***** COMDES-MELT Release Version 1 *****
***** Gasplus Fluid Properties *****
*****
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Helium Axial Compressor 7-7-2014

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 1  
 BLEED = 0.000 DPInc = 6.363 EfDer = 1.000 SH = 0.000000E+00

W act	RPM act	Pt	Tt	POTS	POTH	AeroBl
97.5000	68000.0000	848.4616	90.0000	1.0000	1.0000	0.9500

W Kg/sec	Wdry	H2Ovap_Lb/s	H2Ovap_g/m^3	m^3/min	IWAR	IWC
44.3182	97.5000	0.0000	0.0000	51.1037	0.0000	0.0000

W cor	RPM cor	GAMMA	Cp	R	Blades	THK	ArealUB
0.7036	163247.2188	1.7558	1.3071	386.2500	40	0.0200	8.2122

CFM	SCFM	Al/A*	Areal	A*	AthrRotor	ChokeMargin	AnnularArea
1804.708	0.000	1.5256	7.8016	5.1138	4.6584	0.9109	8.1996

Station 1. ROTOR LEADING EDGE CONDITIONS, STAGE 1

	R1	Stator	Alfa	C1	CU1	Cm1	Mabs	Mrel	U1cor	FlowCoeff
TIP	1.90	0.00	-0.02	555.18	-0.19	555.18	0.41	0.92	2706.66	0.492
MEAN	1.52	0.00	-0.02	555.18	-0.19	555.18	0.41	0.78		
HUB	1.00	0.00	-0.02	555.18	-0.19	555.18	0.41	0.60		

	BetaFlo	BetaBlade	Incid	U1	W1	Ps1	Ts1	TwetBulb1	RH
TIP	63.79	57.50	6.29	1127.48	1256.93	736.11	84.66	90.00	0.00
MEAN	58.36	52.00	6.36	900.93	1058.42	736.11	84.66	90.00	0.00
HUB	46.92	40.50	6.42	593.41	812.77	736.11	84.66	90.00	0.00

Station 2. ROTOR EXIT CONDITIONS, STAGE 1 SOLUTION IS CONVERGED

B2 axial	THK	AeroBl	Blades2	Area2	Area2UB
0.0500	0.0200	0.9500	40	6.5676	7.5545

	R2	C2	Cu2	Cm2	Ao2	Mach2	Chord	AxChord	Rcircle
TIP	1.90	717.45	414.76	585.41	1442.67	0.50	0.54	0.37	2.80
MEAN	1.55	766.92	472.98	603.70	1425.56	0.54	0.49	0.39	1.38
HUB	1.10	853.78	557.75	646.42	1389.68	0.61	0.45	0.41	0.70

	U2	W2	Wu2	MachRel2	DelRCu	Eff2uC	Eff2incC	AvgREff	Wsl/W2
TIP	1127.48	922.32	712.72	0.64	788.41	0.88	0.88		
MEAN	921.22	751.92	448.24	0.53	734.55	0.88	0.87	0.87	1.62
HUB	652.75	653.37	95.00	0.47	613.71	0.85	0.85		

	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
TIP	1149.82	1.36	934.30	104.30	1.16	95.38	104.30	0.00		
MEAN	1125.85	1.33	884.63	103.32	1.15	93.14	103.32	0.00	102.92	1.31
HUB	1070.17	1.26	785.09	101.13	1.12	88.51	101.13	0.00		

	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter
TIP	35.32	50.60	46.50	4.10	0.92	0.36	1.80	0.00037	29
MEAN	38.08	36.59	31.50	5.09	0.92	0.40	2.07	0.00071	37
HUB	40.79	8.36	3.50	4.86	0.92	0.32	2.84	0.00143	37

Station 3. STATOR ROW #1

blockage3	XBladeGap	Vane#	Area3	Tt3	TwetBulb3	MeltFract	Area3UB
0.9500	0.1034	31	6.6080	102.9178	102.9178	0.0000	7.4428

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE

R3m	C3	Cu3	Cm3	Ao3	Mach3	cp 2-3	Stat Ax Chd
1.5582	765.5780	471.2398	603.3595	1422.7319	0.5381	-0.0340	0.4945

Pt3	Ps3	Ts3	RH3	FloAlpha3	VaneAlpha3	Incid3
1115.2797	876.2280	92.7667	0.0000	37.9908	38.5000	-0.5092



Station 4. STATOR / VANED DIFFUSER  
 RCG Cth Cuth Cmth Aoth Machth cp 2-Th Stat Chord  
 1.9000 699.9386 185.8730 674.8076 1435.4510 0.4876 0.1655 0.5237

BlockageTh PtTh PsTh TsTh TwetBulbTh AreaTh w2-Th DiffFact4  
 0.9700 1110.0492 908.9379 94.4328 102.9178 5.9774 0.0212 0.5315

Station 5. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):  
 R4 C4 Cu4 Cm4 Ao4 Mach4 cp 3-4 Stator Gap  
 1.5282 531.9310 0.0000 531.9310 1462.4408 0.3637 0.4711 0.1236

Blockage4 Ps4 Ts4 TwetBulb4 VaneAlpha4 Vane Thk4 w2-4OD cp 2-4  
 0.9800 988.8353 98.0173 102.9178 0.0000 0.0100 0.0314 0.4886

STAGE EXIT CONDITIONS, STAGE 1  
 Eff4 Pt4 PR TR Tt4 Del T Ns Ns nondim  
 0.8467 1107.5037 1.3053 1.1435 102.9178 12.9178 303.9084 2.3559

Del Enthalpy Del\_H/U^2 GHP Reynolds# Specif Humid Area4 Area4UB  
 422698.188 0.333 2328.988 25861678.0 0.00000 7.01773 8.00823

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 2  
 BLEED = 0.000 DPinc = 5.923 EfDer = 0.999 SH = 0.000000E+00

W act RPM act Pt Tt POTS POTH AeroBl  
 97.5000 68000.0000 1107.5037 102.9178 1.0000 1.0000 0.9500

W Kg/sec Wdry H2Ovap\_Lb/s H2Ovap\_g/m^3 m^3/min IWAR IWC  
 44.3182 97.5000 0.0000 0.0000 44.5997 0.0000 0.0000

W cor RPM cor GAMMA Cp R Blades THK ArealUB  
 0.5765 152658.7500 1.7497 1.3026 386.2500 40 0.0200 6.8721

CFM SCFM A1/A\* Areal A\* AthrRotor ChokeMargin AnnularArea  
 1575.023 553896.062 1.5566 6.5285 4.1942 3.8858 0.9265 6.8549

Station 6. ROTOR LEADING EDGE CONDITIONS, STAGE 2

TIP R1 Stator Alfa C1 CU1 Cm1 Mabs Mrel Ulcor FlowCoeff  
 1.90 0.00 -0.02 579.01 -0.20 579.01 0.40 0.87 2531.10 0.514  
 MEAN 1.59 0.00 -0.02 579.01 -0.20 579.01 0.40 0.76  
 HUB 1.20 0.00 -0.02 579.01 -0.20 579.01 0.40 0.63

BetaFlo BetaBlade Incid U1 W1 Psi Ts1 TwetBulb1 RH  
 TIP 62.82 57.00 5.82 1127.48 1267.64 967.75 97.14 102.92 0.00  
 MEAN 58.42 52.50 5.92 941.83 1105.74 967.75 97.14 102.92 0.00  
 HUB 50.78 44.50 6.28 709.13 915.64 967.75 97.14 102.92 0.00

Station 7. ROTOR EXIT CONDITIONS, STAGE 2 SOLUTION IS CONVERGED

B2 axial THK AeroBl Blades2 Area2 Area2UB  
 0.0500 0.0200 0.9500 40 5.6334 6.4514

R2 C2 Cu2 Cm2 Ao2 Mach2 Chord AxChord Rcircle  
 TIP 1.90 745.14 442.52 599.50 1537.08 0.48 0.54 0.38 2.47  
 MEAN 1.61 789.92 496.94 614.02 1523.27 0.52 0.51 0.40 1.39  
 HUB 1.25 876.83 593.78 645.17 1497.83 0.59 0.47 0.42 0.74

U2 W2 Wu2 MachRel2 DelRCu Eff2uC Eff2incC AvgREff Ws1/W2  
 TIP 1127.48 910.26 684.96 0.59 841.17 0.88 0.88  
 MEAN 954.31 765.65 457.38 0.50 799.48 0.88 0.87 0.87 1.63  
 HUB 741.76 661.93 147.98 0.44 742.47 0.85 0.85

Pt2 PR Ps2 Tt2 TR Ts2 TwetBulb2 RH Tt2avg PR2avg  
 TIP 1475.49 1.33 1211.61 118.23 1.15 108.65 118.23 0.00  
 MEAN 1453.69 1.31 1161.79 117.47 1.14 106.71 117.47 0.00 117.37 1.31  
 HUB 1417.43 1.28 1069.07 116.43 1.13 103.18 116.43 0.00

Alfa2 Beta FLO Beta BLADE Deviat Slip F. DiffFct Solidity Convergence Iter  
 TIP 36.43 48.81 44.50 4.31 0.92 0.38 1.80 0.00022 27  
 MEAN 38.98 36.68 31.50 5.18 0.92 0.42 2.04 0.00095 37  
 HUB 42.62 12.92 7.50 5.42 0.92 0.41 2.52 0.00051 35

Station 8. STATOR ROW #1  
 blockage3 XBladeGap Vane# Area3 Tt3 TwetBulb3 MeltFract Area3UB  
 1.0000 0.1062 33 5.9601 117.3750 117.3750 0.0000 6.3847

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE  
 R3m C3 Cu3 Cm3 Ao3 Mach3 cp 2-3 Stat Ax Chd  
 1.6115 760.1998 495.9160 576.1693 1528.2654 0.4974 0.0536 0.4616

Pt3 Ps3 Ts3 RH3 FloAlpha3 VaneAlpha3 Incid3  
 1448.8711 1177.9467 107.4121 0.0000 40.7190 40.5000 0.2190

Station 9. STATOR / VANED DIFFUSER  
 RCG Cth Cuth Cmth Aoth Machth cp 2-Th Stat Chord  
 1.6115 691.5704 192.9420 664.1107 1540.4365 0.4489 0.2320 0.4920

BlockageTh PtTh PsTh TsTh TwetBulbTh AreaTh w2-Th DiffFact4  
 0.9700 1442.9897 1217.4164 109.1297 117.3750 5.2404 0.0195 0.4735

Station 10. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):  
 R4 C4 Cu4 Cm4 Ao4 Mach4 cp 3-4 Stator Gap  
 1.6082 582.5651 0.0000 582.5651 1557.2439 0.3741 0.3674 0.1154

Blockage4 Ps4 Ts4 TwetBulb4 VaneAlpha4 Vane Thk4 w2-4OD cp 2-4  
 0.9800 1277.4941 111.5241 117.3750 0.0000 0.0100 0.0314 0.4314

STAGE EXIT CONDITIONS, STAGE 2  
 Eff4 Pt4 PR TR Tt4 Del T Ns Ns nondim  
 0.8462 1439.4124 1.2997 1.1405 117.3750 14.4571 261.5940 2.0279

Del Enthalpy Del\_H/U^2 GHP Reynolds# Specif Humid Area4 Area4UB  
 471451.438 0.371 2597.609 21319290.0 0.00000 5.64337 6.43241

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 3  
 BLEED = 0.000 DPinc = 6.112 EfDer = 0.999 SH = 0.000000E+00

W act RPM act Pt Tt POTS POTH AeroBl  
 97.5000 68000.0000 1439.4124 117.3750 1.0000 1.0000 0.9500

W Kg/sec Wdry H2Ovap\_Lb/s H2Ovap\_g/m^3 m^3/min IWAR IWC  
 44.3182 97.5000 0.0000 0.0000 38.7492 0.0000 0.0000

W cor RPM cor GAMMA Cp R Blades THK ArealUB  
 0.4429 152863.3594 1.7582 1.3089 386.2500 40 0.0200 5.9655

CFM SCFM A1/A\* Areal A\* AthrRotor ChokeMargin AnnularArea  
 1368.416 553896.062 1.6470 5.6672 3.4409 3.3467 0.9726 5.9441

Station 11. ROTOR LEADING EDGE CONDITIONS, STAGE 3

	R1	Stator	Alfa	C1	CU1	Cm1	Mabs	Mrel	U1cor	FlowCoeff
TIP	1.90	0.00	-0.02	579.51	-0.20	579.51	0.37	0.81	2534.50	0.514
MEAN	1.63	0.00	-0.02	579.51	-0.20	579.51	0.37	0.72		
HUB	1.31	0.00	-0.02	579.51	-0.20	579.51	0.37	0.62		

	BetaFlo	BetaBlade	Incid	U1	W1	Ps1	Ts1	TwetBulb1	RH
TIP	62.80	56.50	6.30	1127.48	1267.87	1279.11	111.55	117.37	0.00
MEAN	59.11	53.00	6.11	968.55	1128.85	1279.11	111.55	117.37	0.00
HUB	53.32	47.00	6.32	777.79	970.10	1279.11	111.55	117.37	0.00

Station 12. ROTOR EXIT CONDITIONS, STAGE 3 SOLUTION IS CONVERGED  
 B2 axial THK AeroBl Blades2 Area2 Area2UB  
 0.0500 0.0200 0.9500 40 4.9371 5.6388

	R2	C2	Cu2	Cm2	Ao2	Mach2	Chord	AxChord	Rcircle
TIP	1.90	758.13	459.19	603.25	1640.76	0.46	0.54	0.39	2.37
MEAN	1.65	803.69	518.68	613.91	1630.37	0.49	0.52	0.41	1.39
HUB	1.35	909.99	649.06	637.82	1611.67	0.56	0.49	0.43	0.72

	U2	W2	Wu2	MachRel2	DelRCu	Eff2uC	Eff2incC	AvgREff	Ws1/W2
TIP	1127.48	900.29	668.29	0.55	872.84	0.88	0.88		
MEAN	978.01	766.72	459.33	0.47	855.16	0.88	0.87	0.87	1.64
HUB	801.11	655.69	152.04	0.41	876.50	0.85	0.85		

	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
TIP	1866.11	1.30	1557.96	133.18	1.13	123.21	133.18	0.00		
MEAN	1854.30	1.29	1511.60	132.86	1.13	121.66	132.86	0.00	133.10	1.29
HUB	1852.37	1.29	1421.76	133.25	1.14	118.88	133.25	0.00		

	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter
TIP	37.28	47.93	43.50	4.43	0.92	0.39	1.80	0.00028	24
MEAN	40.19	36.80	31.50	5.30	0.92	0.44	2.02	0.00054	35
HUB	45.50	13.41	7.50	5.91	0.92	0.47	2.36	0.00179	35

Station 13. STATOR ROW #1

blockage3	XBladeGap	Vane#	Area3	Tt3	TwetBulb3	MeltFract	Area3UB
1.0000	0.1079	51	5.0174	133.0993	133.0993	0.0000	5.5728

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE

R3m	C3	Cu3	Cm3	Ao3	Mach3	cp 2-3	Stat Ax Chd
1.6513	794.1422	517.6735	602.2258	1633.7197	0.4861	0.0302	0.2987

Pt3	Ps3	Ts3	RH3	FloAlpha3	VaneAlpha3	Incid3
1857.5942	1522.4895	122.1572	0.0000	40.6824	40.5000	0.1824

Station 14. STATOR / VANED DIFFUSER

RCG	Cth	Cuth	Cmth	Aoth	Machth	cp 2-Th	Stat Chord
1.6513	766.4976	213.8460	736.0628	1638.7174	0.4677	0.1148	0.3183

BlockageTh	PtTh	PsTh	TsTh	TwetBulbTh	AreaTh	w2-Th	DiffFact4
0.9700	1850.7207	1538.4973	122.9058	133.0993	4.2137	0.0191	0.3926

Station 15. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):

R4	C4	Cu4	Cm4	Ao4	Mach4	cp 3-4	Stator Gap
1.6642	672.7041	0.0000	672.7041	1654.2573	0.4067	0.2417	0.0747

Blockage4	Ps4	Ts4	TwetBulb4	VaneAlpha4	Vane Thk4	w2-4OD	cp 2-4
0.9800	1603.4835	125.2478	133.0993	0.0000	0.0100	0.0314	0.2951

STAGE EXIT CONDITIONS, STAGE 3

Eff4	Pt4	PR	TR	Tt4	Del T	Ns	Ns nondim
0.8459	1846.2820	1.2827	1.1340	133.0993	15.7244	228.1183	1.7684

Del Enthalpy	Del H/U^2	GHP	Reynolds#	Specif Humid	Area4	Area4UB
515245.969	0.405	2838.909	21305586.0	0.00000	4.37275	5.28088

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 4

BLEED = 0.000 DPInc = 6.555 EfDer = 1.000 SH = 0.000000E+00

W act	RPM act	Pt	Tt	POTS	POTH	AeroBl
97.5000	68000.0000	1846.2820	133.0993	1.0000	1.0000	0.9500

W Kg/sec	Wdry	H2Ovap_Lb/s	H2Ovap_g/m^3	m^3/min	IWAR	IWC
44.3182	97.5000	0.0000	0.0000	33.9944	0.0000	0.0000

W cor	RPM cor	GAMMA	Cp	R	Blades	THK	ArealUB
0.3443	153301.2188	1.7625	1.3122	386.2500	40	0.0200	5.2095

CFM	SCFM	A1/A*	Areal	A*	AthrRotor	ChokeMargin	AnnularArea
1200.499	553896.062	1.7338	4.9490	2.8544	2.9338	1.0278	5.1836

Station 16. ROTOR LEADING EDGE CONDITIONS, STAGE 4

	R1	Stator	Alfa	Cl	CU1	Cm1	Mabs	Mrel	U1cor	FlowCoeff
TIP	1.90	0.00	-0.02	582.18	-0.20	582.18	0.35	0.76	2541.76	0.516
MEAN	1.67	0.00	-0.02	582.18	-0.20	582.18	0.35	0.69		
HUB	1.40	0.00	-0.02	582.18	-0.20	582.18	0.35	0.61		

	BetaFlo	BetaBlade	Incid	U1	W1	Ps1	Ts1	TwetBulb1	RH
TIP	62.69	56.50	6.19	1127.48	1269.09	1662.59	127.20	133.10	0.00
MEAN	59.55	53.00	6.55	990.30	1148.93	1662.59	127.20	133.10	0.00
HUB	54.99	48.00	6.99	830.78	1014.62	1662.59	127.20	133.10	0.00

Station 17. ROTOR EXIT CONDITIONS, STAGE 4 SOLUTION IS CONVERGED

B2 axial	THK	AeroBl	Blades2	Area2	Area2UB
0.0500	0.0200	0.9500	40	4.3367	4.9429

	R2	C2	Cu2	Cm2	Ao2	Mach2	Chord	AxChord	Rcircle
TIP	1.90	794.87	513.31	606.90	1749.42	0.45	0.54	0.40	1.93
MEAN	1.68	835.83	564.14	616.73	1739.10	0.48	0.53	0.42	1.29
HUB	1.43	941.00	692.71	636.90	1723.32	0.55	0.50	0.44	0.73

	U2	W2	Wu2	MachRel2	DelRCu	Eff2uC	Eff2incC	AvgREff	Ws1/W2
TIP	1127.48	863.44	614.17	0.49	975.68	0.88	0.88		
MEAN	997.87	753.98	433.73	0.43	948.99	0.88	0.87	0.87	1.67
HUB	848.70	655.72	155.99	0.38	990.99	0.85	0.85		

	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
TIP	2382.00	1.29	1999.34	150.73	1.13	139.73	150.73	0.00		
MEAN	2363.15	1.28	1944.31	150.24	1.13	138.08	150.24	0.00	150.66	1.28
HUB	2371.14	1.28	1848.75	151.00	1.13	135.59	151.00	0.00		

	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter
TIP	40.22	45.34	40.50	4.84	0.92	0.43	1.80	0.00059	23
MEAN	42.45	35.12	29.50	5.62	0.92	0.47	2.01	0.00126	36
HUB	47.40	13.76	7.50	6.26	0.92	0.50	2.29	0.00005	33

Station 18. STATOR ROW #1

blockage3	XBladeGap	Vane#	Area3	Tt3	TwetBulb3	MeltFract	Area3UB
0.9500	0.1112	40	4.2879	150.6583	150.6583	0.0000	4.8866

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE

R3m	C3	Cu3	Cm3	Ao3	Mach3	cp 2-3	Stat Ax Chd
1.6843	840.3726	563.2435	623.6849	1740.8657	0.4827	0.0094	0.3789

Pt3	Ps3	Ts3	RH3	FloAlpha3	VaneAlpha3	Incid3
2372.0977	1948.4457	138.3655	0.0000	42.0849	42.0000	0.0849

Station 19. STATOR / VANED DIFFUSER

RCG	Cth	Cuth	Cmth	Aoth	Machth	cp 2-Th	Stat Chord
1.6843	789.7696	228.2685	756.0618	1749.8751	0.4513	0.1307	0.4059

BlockageTh	PtTh	PsTh	TsTh	TwetBulbTh	AreaTh	w2-Th	DiffFact4
0.9700	2363.7437	1988.4733	139.8014	150.6583	3.6102	0.0189	0.4816

Station 20. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):

R4	C4	Cu4	Cm4	Ao4	Mach4	cp 3-4	Stator Gap
1.6954	642.7379	0.0000	642.7379	1772.6711	0.3626	0.3723	0.0947

Blockage4	Ps4	Ts4	TwetBulb4	VaneAlpha4	Vane Thk4	w2-40D	cp 2-4
0.9800	2106.1824	143.4676	150.6583	0.0000	0.0100	0.0314	0.3974

STAGE EXIT CONDITIONS, STAGE 4

Eff4	Pt4	PR	TR	Tt4	Del T	Ns	Ns nondim
0.8466	2358.2529	1.2773	1.1319	150.6583	17.5590	196.3234	1.5219

Del Enthalpy	Del_H/U^2	GHP	Reynolds#	Specif Humid	Area4	Area4UB
576803.312	0.454	3178.079	21184304.0	0.00000	3.99114	4.62250

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 5  
 BLEED = 0.000 DPInc = 5.983 EfDer = 0.999 SH = 0.000000E+00

W act	RPM act	Pt	Tt	POTS	POTH	AeroBl
97.5000	68000.0000	2358.2529	150.6583	1.0000	1.0000	0.9500

W Kg/sec	Wdry	H2Ovap_Lb/s	H2Ovap_g/m^3	m^3/min	IWAR	IWC
44.3182	97.5000	0.0000	0.0000	29.9175	0.0000	0.0000

W cor	RPM cor	GAMMA	Cp	R	Blades	THK	ArealUB
0.2693	153439.4844	1.7595	1.3101	386.2500	40	0.0200	4.5822

CFM	SCFM	Al/A*	Areal	A*	AthrRotor	ChokeMargin	AnnularArea
1056.525	553896.062	1.8299	4.3531	2.3789	2.5195	1.0591	4.5516

Station 21. ROTOR LEADING EDGE CONDITIONS, STAGE 5

	R1	Stator	Alfa	C1	CU1	Cm1	Mabs	Mrel	U1cor	FlowCoeff
TIP	1.90	0.00	-0.02	582.49	-0.20	582.49	0.33	0.71	2544.05	0.517
MEAN	1.70	0.00	-0.02	582.49	-0.20	582.49	0.33	0.65		
HUB	1.47	0.00	-0.02	582.49	-0.20	582.49	0.33	0.59		

	BetaFlo	BetaBlade	Incid	U1	W1	Ps1	Ts1	TwetBulb1	RH
TIP	62.68	56.50	6.18	1127.48	1269.24	2150.05	144.77	150.66	0.00
MEAN	59.98	54.00	5.98	1008.03	1164.40	2150.05	144.77	150.66	0.00
HUB	56.27	50.00	6.27	872.37	1049.14	2150.05	144.77	150.66	0.00

Station 22. ROTOR EXIT CONDITIONS, STAGE 5 SOLUTION IS CONVERGED

B2 axial	THK	AeroBl	Blades2	Area2	Area2UB
0.0500	0.0200	0.9500	40	3.7842	4.3058

	R2	C2	Cu2	Cm2	Ao2	Mach2	Chord	AxChord	Rcircle
TIP	1.90	797.98	501.62	620.60	1851.92	0.43	0.54	0.40	1.93
MEAN	1.71	850.49	574.77	626.87	1846.32	0.46	0.53	0.42	1.25
HUB	1.50	972.82	729.79	643.26	1835.22	0.53	0.51	0.44	0.70

	U2	W2	Wu2	MachRel2	DelRCu	Eff2uC	Eff2incC	AvgREff	Wsl/W2
TIP	1127.48	881.38	625.86	0.48	953.47	0.88	0.88		
MEAN	1015.76	766.44	440.98	0.42	984.19	0.88	0.87	0.87	1.65
HUB	890.12	662.94	160.32	0.36	1094.99	0.85	0.85		

	Pt2	PR	Ps2	Tt2	TR	Ts2	TwetBulb2	RH	Tt2avg	PR2avg
TIP	2945.34	1.25	2515.29	167.91	1.11	156.85	167.91	0.00		
MEAN	2961.83	1.26	2475.09	168.47	1.12	155.91	168.47	0.00	168.95	1.26
HUB	3014.02	1.28	2383.09	170.47	1.13	154.04	170.47	0.00		

	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter
TIP	38.95	45.24	40.50	4.74	0.92	0.42	1.80	0.00090	23
MEAN	42.52	35.13	29.50	5.63	0.92	0.47	1.99	0.00130	33
HUB	48.61	14.00	7.50	6.50	0.92	0.53	2.20	0.00203	32

Station 23. STATOR ROW #1

blockage3	XBladeGap	Vane#	Area3	Tt3	TwetBulb3	MeltFract	Area3UB
0.9500	0.1112	40	3.7842	168.9509	168.9509	0.0000	4.3058

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE

R3m	C3	Cu3	Cm3	Ao3	Mach3	cp 2-3	Stat Ax Chd
1.7117	849.7291	574.7733	625.8397	1849.3101	0.4595	0.0234	0.3783

Pt3	Ps3	Ts3	RH3	FloAlpha3	VaneAlpha3	Incid3
2973.7288	2487.1890	156.4113	0.0000	42.5645	42.5000	0.0645

Station 24. STATOR / VANED DIFFUSER

RCG	Cth	Cuth	Cmth	Aoth	Machth	cp 2-Th	Stat Chord
1.7117	799.4977	233.7505	764.5635	1857.7960	0.4303	0.1449	0.4059

BlockageTh	PtTh	PsTh	TsTh	TwetBulbTh	AreaTh	w2-Th	DiffFact4
0.9700	2964.4329	2532.5715	157.8501	168.9509	3.1650	0.0180	0.5269

Station 25. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):

R4	C4	Cu4	Cm4	Ao4	Mach4	cp 3-4	Stator Gap
1.7117	613.3331	0.0000	613.3331	1884.4843	0.3255	0.4360	0.0946

Blockage4	Ps4	Ts4	TwetBulb4	VaneAlpha4	Vane Thk4	w2-40D	cp 2-4
0.9800	2699.3140	162.4179	168.9509	0.0000	0.0100	0.0314	0.4681

STAGE EXIT CONDITIONS, STAGE 5

Eff4	Pt4	PR	TR	Tt4	Del T	Ns	Ns nondim
0.8456	2957.5439	1.2541	1.1214	168.9509	18.2925	178.8205	1.3862

Del Enthalpy	Del_H/U^2	GHP	Reynolds#	Specif Humid	Area4	Area4UB
599946.812	0.472	3305.595	20689816.0	0.00000	3.69451	4.27257

OVERALL EXIT CONDITIONS; ALL 5 STAGES

Del Enthal	DelHT/U1^2	GHP	MassFloSlcor	OPR	Efficiency	RotorlInc	TR	AxHubLen
2586145.75	2.0344	14249.1807	0.704	3.4857	0.81239	6.3628	1.8772	5.215

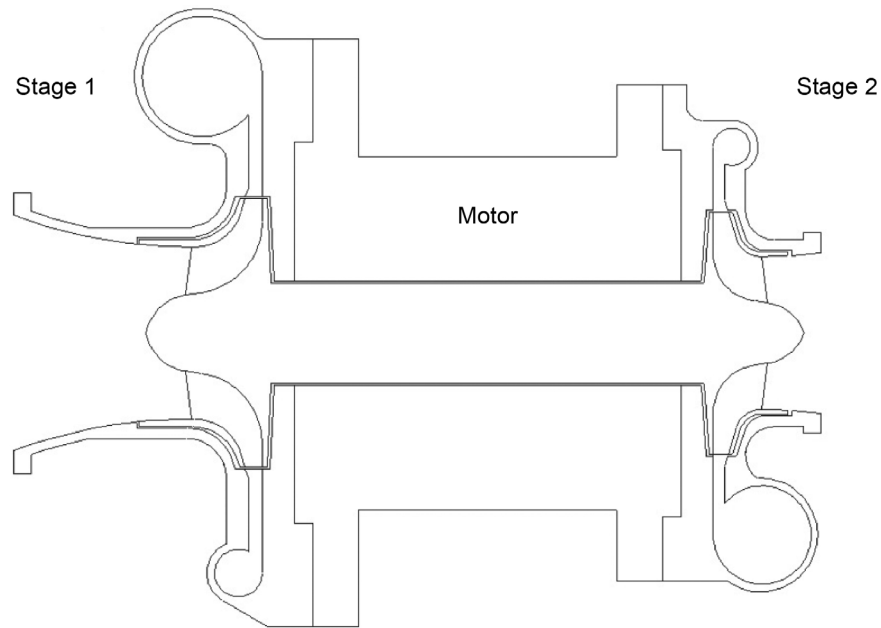


Figure 29.—Flow path for the two-stage centrifugal carbon dioxide compressor.

## 7.6 Example Case (6): Two-Stage Carbon Dioxide Compressor

The input and output files in this section are those used for the analysis of a two-stage carbon dioxide compressor. A cross section of the compressor is shown in Figure 29.

### 7.6.1 Input File

comdes.dat

```

Centrifugal Compressor for CO2
3.08      0.0773    383.6      1666.7      Wact      Ptl      Ttl      RPM
7         2           0           0.00       Fluid    Stages   IDUMMY   DUMMY
0.00     46.900     000.00     0.0        DropDia  InDist   RHinlet  IWC      ***icing***
0.077    0.0        383.6      0.0        Psin     DUMMY    Tsin     DUMMY    ***icing***
100.00   0.0        00000.     0          Vin      DUMMY    Altitude IDUMMY   ***icing***
32.7     16.2       51.0       51.0       1 Rltip   Rlhub    R2tip    R2hub
0.9000   6.70       23         0.1700    1.1     Blaxial  B2axial  NBLAD   THK      CFS
0.0      0.0000     0.000     1.0000    1.1     StatorT  StatorM  StatorH  FSGV    CFMid
0.9800   0.9500     1.1000    0.9000    1.1     AeroB1   AeroB2   POTS     POTH    CFHub
0.89     0.89       0.89      0.0000    EfficiT  EfficiM  EfficiH  BLEED
0.00     0.00       0.00      0.00      DUMMY    DUMMY    DUMMY    DUMMY
0.880    0.16       1.35      0.00      SlipF    DiffLoss Solidity  RPM2
55.8000  52.4       42.70     2.0       BetaBlade1T BetaBlade1M BetaBlade1H expo3
37.00    37.0       37.00     0         BetaBlade2T BetaBlade2M BetaBlade2H stat2
56.00    56.00     5.2       65.0      R3s      R3h      B3X      BetB3
21       0.10       1.00      0.02     Blan3    Blathk3  Blok3    Wbar23
0.0      2000.0     0.97      0.10     RCG     Athrt    BlokTH   Wbar2T
83.0     83.0       5.2       30.0     R4S     R4H      B4X      BetB4
21       0.1        0.95      1.40     Blan4    Blathk4  Blok4    SolidStat
29.0     11.5       46.0      46.0     2 Rltip   Rlhub    R2tip    R2hub
0.90     5.8       23         0.1000   1.0     Blaxial  B2axial  NBLAD   THK      CFS
0.0      0.0000     0.000     1.0000   1.0     StatorT  StatorM  StatorH  FSGV    CFMid
0.9800   0.9500     1.1000    0.9000   1.0     AeroB1   AeroB2   POTS     POTH    CFHub
0.89     0.89       0.89      0.0000   EfficiT  EfficiM  EfficiH  BLEED
0.00     0.00       0.00      0.00     DUMMY    DUMMY    DUMMY    DUMMY
0.880    0.16       1.35      0.00     SlipF    DiffLoss Solidity  RPM2
61.0000  56.0       42.5000   2.0     BetaBlade1T BetaBlade1M BetaBlade1H expo3
37.00    37.0       37.00     0         BetaBlade2T BetaBlade2M BetaBlade2H stat2
50.6     50.6       4.5       65.0     R3s      R3h      B3X      BetB3
51       0.07       1.00      0.05     Blan3    Blathk3  Blok3    Wbar23
0.0      1600.0     0.97      0.10     RCG     Athrt    BlokTH   Wbar2T
75.00    75.00     4.5       30.0     R4S     R4H      B4X      BetB4

```

51	0.07	0.95	1.40	Blan4	Blathk4	Blok4	SolidStat
30.0	26.0	0.83	2.634	R7S	R7h	B7X	Length67
4	0.1	0.98	0.00	Blan7	Blath7	Blok7	Wbar67
30.0	26.0	0.0	10.53	R8S	R8h	B8X	Length78
4	0.1	0.98	0.00	Blan8	Blath8	Blok8	Wbar78

## 7.6.2 Output File

### comdes.out

```

*****
***** AXIAL & CENTRIFUGAL COMPRESSOR BLADE DESIGN 1-D ****
***** COMDES-MELT Release Version 1 *****
***** Gasplus Fluid Properties *****
*****

Centrifugal Compressor for CO2
Fluid = CO2 RG 7 35.11364

COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 1
BLEED = 0.000 DPInc = 6.604 EfDer = 1.000 SH = 0.000000E+00

W act RPM act Pt Tt POTS POTH AeroBl
3.0800 1666.7000 0.0773 383.6000 1.1000 0.9000 0.9800

W Kg/sec Wdry H2Ovap_Lb/s H2Ovap_g/m^3 m^3/min IWAR IWC
1.4000 3.0800 0.0000 0.0000 6619.7456 0.0000 0.0000

W cor RPM cor GAMMA Cp R Blades THK ArealUB
503.6974 1938.1002 1.3317 0.1813 35.1136 23 0.1700 2538.5620

CFM SCFM A1/A* Areal A* AthrRotor ChokeMargin AnnularArea
233774.094 0.000 2.0535 2487.7908 1211.5072 1635.4521 1.3499 2534.7939

Station 1. ROTOR LEADING EDGE CONDITIONS, STAGE 1

TIP R1 Stator Alfa C1 CU1 Cm1 Mabs Mrel Ulcor FlowCoeff
25.80 0.00 -0.02 248.08 -0.09 248.08 0.33 0.71 553.04 0.474
MEAN 25.80 0.00 -0.02 225.52 -0.08 225.52 0.30 0.58
HUB 16.20 0.00 -0.02 202.97 -0.07 202.97 0.27 0.41

BetaFlo BetaBlade Incid U1 W1 Ps1 Ts1 TwetBulb1 RH
TIP 62.46 55.80 6.66 475.61 536.50 0.07 376.82 383.60 0.00
MEAN 59.00 52.40 6.60 375.32 437.93 0.07 377.99 383.60 0.00
HUB 49.27 42.70 6.57 235.62 311.05 0.07 379.06 383.60 0.00

Station 2. ROTOR EXIT CONDITIONS, STAGE 1 SOLUTION IS CONVERGED
B2 axial THK AeroBl Blades2 Area2 Area2UB
6.7000 0.1700 0.9500 23 2014.7291 2146.9646

R2 C2 Cu2 Cm2 Ao2 Mach2 Chord AxChord Rcircle
TIP 51.00 540.65 502.56 199.33 807.61 0.67 18.81 14.59 57.58
MEAN 51.00 540.65 502.56 199.34 807.61 0.67 20.28 15.22 75.66
HUB 51.00 540.65 502.56 199.34 807.61 0.67 21.12 16.21 212.37

U2 W2 Wu2 MachRel2 DelRCu Eff2uC Eff2incC AvgREff Ws1/W2
TIP 741.78 311.38 239.22 0.39 25633.45 0.89 0.89
MEAN 741.78 311.39 239.22 0.39 25632.51 0.89 0.89 0.89 1.72
HUB 741.78 311.39 239.22 0.39 25631.52 0.89 0.89

Pt2 PR Ps2 Tt2 TR Ts2 TwetBulb2 RH Tt2avg PR2avg
TIP 0.16 2.01 0.12 465.74 1.21 433.52 465.74 0.00
MEAN 0.16 2.01 0.12 465.74 1.21 433.51 465.74 0.00 465.74 2.01
HUB 0.16 2.01 0.12 465.74 1.21 433.51 465.74 0.00

Alfa2 Beta FLO Beta BLADE Deviat Slip F. DiffFct Solidity Convergence Iter
TIP 68.37 50.20 37.00 13.20 0.88 0.84 1.35 0.00052 87
MEAN 68.36 50.20 37.00 13.20 0.88 0.55 2.88 0.00047 99
HUB 68.36 50.20 37.00 13.20 0.88 0.26 4.77 0.00058 106

Station 3. STATOR ROW #1
blockage3 XBladeGap Vane# Area3 Tt3 TwetBulb3 MeltFract Area3UB
1.0000 4.0534 21 1818.7435 465.7393 465.7393 0.0000 1829.6636

```

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE  
 R3m C3 Cu3 Cm3 Ao3 Mach3 cp 2-3 Stat Ax Chd  
 56.0000 506.0438 457.6877 215.8758 811.3214 0.6237 0.0968 15.8475  
  
 Pt3 Ps3 Ts3 RH3 FloAlpha3 VaneAlpha3 Incid3  
 0.1549 0.1205 437.5070 0.0000 64.7483 65.0000 -0.2517  
  
 Station 4. STATOR / VANED DIFFUSER  
 RCG Cth Cuth Cmth Aoth Machth cp 2-Th Stat Chord  
 56.0000 253.1019 175.8194 182.0663 830.7183 0.3047 0.6087 23.4572  
  
 BlockageTh PtTh PsTh TsTh TwetBulbTh AreaTh w2-Th DiffFact4  
 0.9700 0.1493 0.1404 458.6768 465.7393 2000.0000 0.1632 0.8896  
  
 Station 5. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):  
 R4 C4 Cu4 Cm4 Ao4 Mach4 cp 3-4 Stator Gap  
 83.0000 154.5870 77.2935 133.8763 834.7183 0.1852 0.7354 3.9619  
  
 Blockage4 Ps4 Ts4 TwetBulb4 VaneAlpha4 Vane Thk4 w2-40D cp 2-4  
 0.9500 0.1458 463.1047 465.7393 30.0000 0.1000 0.1674 0.7463  
  
 STAGE EXIT CONDITIONS, STAGE 1  
 Eff4 Pt4 PR TR Tt4 Del T Ns Ns nondim  
 0.8309 0.1492 1.9297 1.2141 465.7393 82.1393 93.1419 0.7220  
  
 Del Enthalpy Del\_H/U^2 GHP Reynolds# Specif Humid Area4 Area4UB  
 372864.906 0.678 64.898 34129.4 0.00000 2565.85767 2711.82251  
  
 Fluid = CO2 RG 7 35.11364  
  
 COMPRESSOR INLET CONDITIONS - ANALYSIS, STAGE 2  
 BLEED = 0.000 DPInc = 7.877 EfDer = 0.999 SH = 0.000000E+00  
  
 W act RPM act Pt Tt POTS POTH AeroBl  
 3.0800 1666.7000 0.1492 465.7393 1.1000 0.9000 0.9800  
  
 W Kg/sec Wdry H2Ovap\_Lb/s H2Ovap\_g/m^3 m^3/min IWAR IWC  
 1.4000 3.0800 0.0000 0.0000 4056.8645 0.0000 0.0000  
  
 W cor RPM cor GAMMA Cp R Blades THK ArealUB  
 287.6216 1758.9120 1.3060 0.1928 35.1136 23 0.1000 2229.5464  
  
 CFM SCFM Al/A\* Areal A\* AthrRotor ChokeMargin AnnularArea  
 143266.812 1590.676 3.1369 2184.9553 696.5395 1328.1255 1.9067 2226.6038  
  
 Station 6. ROTOR LEADING EDGE CONDITIONS, STAGE 2  
 R1 Stator Alfa C1 CU1 Cml Mabs Mrel Ulcor FlowCoeff  
 TIP 29.00 0.00 -0.02 173.10 -0.06 173.10 0.21 0.55 445.12 0.373  
 MEAN 22.06 0.00 -0.02 157.37 -0.05 157.37 0.19 0.43  
 HUB 11.50 0.00 -0.02 141.63 -0.05 141.63 0.17 0.27  
  
 BetaFlo BetaBlade Incid U1 W1 Ps1 Ts1 TwetBulb1 RH  
 TIP 67.69 61.00 6.69 421.80 455.99 0.14 462.63 465.74 0.00  
 MEAN 63.88 56.00 7.88 320.85 357.41 0.15 463.17 465.74 0.00  
 HUB 49.75 42.50 7.25 167.26 219.21 0.15 463.66 465.74 0.00  
  
 Station 7. ROTOR EXIT CONDITIONS, STAGE 2 SOLUTION IS CONVERGED  
 B2 axial THK AeroBl Blades2 Area2 Area2UB  
 5.8000 0.1000 0.9500 23 1579.8632 1676.3539  
  
 R2 C2 Cu2 Cm2 Ao2 Mach2 Chord AxChord Rcircle  
 TIP 46.00 489.43 457.26 174.52 862.60 0.57 16.96 12.52 40.80  
 MEAN 46.00 489.43 457.26 174.52 862.60 0.57 17.97 13.06 54.42  
 HUB 46.00 489.43 457.26 174.53 862.60 0.57 18.10 13.91 188.58  
  
 U2 W2 Wu2 MachRel2 DelRCu Eff2uC Eff2incC AvgREff Wsl/W2  
 TIP 669.06 274.44 211.80 0.32 21035.65 0.89 0.89  
 MEAN 669.06 274.44 211.80 0.32 21035.14 0.89 0.89 0.89 1.66  
 HUB 669.06 274.44 211.80 0.32 21034.39 0.89 0.89  
  
 Pt2 PR Ps2 Tt2 TR Ts2 TwetBulb2 RH Tt2avg PR2avg  
 TIP 0.24 1.63 0.20 529.14 1.14 504.30 529.14 0.00  
 MEAN 0.24 1.63 0.20 529.14 1.14 504.30 529.14 0.00 529.14 1.63  
 HUB 0.24 1.63 0.20 529.14 1.14 504.30 529.14 0.00



	Alfa2	Beta FLO	Beta BLADE	Deviat	Slip F.	DiffFct	Solidity	Convergence	Iter
TIP	69.11	50.51	37.00	13.51	0.88	0.85	1.35	0.00046	66
MEAN	69.11	50.51	37.00	13.51	0.88	0.52	2.98	0.00085	72
HUB	69.11	50.51	37.00	13.51	0.88	0.04	5.76	0.00064	78

Station 8. STATOR ROW #1

blockage3	XBladeGap	Vane#	Area3	Tt3	TwetBulb3	MeltFract	Area3UB
1.0000	3.4781	51	1414.6163	529.1428	529.1428	0.0000	1430.6813

ROTOR-STATOR GAP EXIT: STATOR LEADING EDGE

R3m	C3	Cu3	Cm3	Ao3	Mach3	cp 2-3	Stat Ax Chd
50.6000	458.2597	415.6907	192.8813	865.2158	0.5296	0.0732	5.8962

Pt3	Ps3	Ts3	RH3	FloAlpha3	VaneAlpha3	Incid3
0.2406	0.2011	507.3655	0.0000	65.1086	65.0000	0.1086

Station 9. STATOR / VANED DIFFUSER

RCG	Cth	Cuth	Cmth	Aoth	Machth	cp 2-Th	Stat Chord
50.6000	223.7194	155.4086	160.9303	879.2451	0.2544	0.6452	8.7275

BlockageTh	PtTh	PsTh	TsTh	TwetBulbTh	AreaTh	w2-Th	DiffFact4
0.9700	0.2366	0.2269	523.9525	529.1428	1600.0000	0.1385	0.8843

Station 10. STATOR / VANED DIFFUSER EXIT (TRAILING EDGE):

R4	C4	Cu4	Cm4	Ao4	Mach4	cp 3-4	Stator Gap
75.0000	142.2885	71.1442	123.2254	881.8345	0.1614	0.7660	1.4740

Blockage4	Ps4	Ts4	TwetBulb4	VaneAlpha4	Vane Thk4	w2-4OD	cp 2-4
0.9500	0.2314	527.0432	529.1428	30.0000	0.0700	0.1673	0.7448

STAGE EXIT CONDITIONS, STAGE 2

Eff4	Pt4	PR	TR	Tt4	Del T	Ns	Ns nondim
0.8282	0.2353	1.5776	1.1361	529.1428	63.4035	84.5678	0.6556

Del Enthalpy	Del_H/U^2	GHP	Reynolds#	Specif Humid	Area4	Area4UB
305988.000	0.684	53.258	33400.4	0.00000	1999.28467	2120.57495

OVERALL EXIT CONDITIONS; ALL 2 STAGES

Del Enthal	DelHT/U1^2	GHP	MassFloSlcor	OPR	Efficiency	RotorlInc	TR	AxHubLen
678852.88	3.0010	118.1568	503.697	3.0443	0.81408	6.6040	1.3794	64.837



## Appendix—Nomenclature

$A$	inlet area, in <sup>2</sup>
$a$	speed of sound, ft/s
$B$	edge sweep, in.
$C$	absolute velocity, ft/s
$\bar{C}$	chord, in.
$c_P$	specific heat at constant pressure, Btu/(lbm·R)
$DF$	diffusion factor, dimensionless
$D_v$	diffusion coefficient of water vapor into air, ft <sup>2</sup> /s
$d$	ice particle size (diameter), m
$F$	molar flow rates, mol/s
$g_c$	unit conversion, 32.2 lbm·ft/(lbf·s <sup>2</sup> )
$\Delta H$	enthalpy change, ft <sup>2</sup> /s <sup>2</sup>
$h$	convective heat transfer coefficient, Btu/(h·ft <sup>2</sup> ·R)
$h_m$	mass transfer coefficient, ft/s
$IWC$	ice water content, kg/m <sup>3</sup>
$i$	incidence angle, deg
$J$	units conversion, 778.28 ft·lbf/Btu
$k$	thermal conductivity, Btu/(h·ft·R)
$L$	latent heat, Btu/lbm
$L_s$	latent heat of sublimation, Btu/lbm
$L_v$	latent heat of evaporation, Btu/lbm
$Le$	Lewis number, dimensionless
$M$	Mach number, dimensionless
$M_a$	molecular weight of air
$M_w$	molecular weight of water vapor
$M_{wet}$	molecular weight of air and water mixture
$m$	mass flow rate, lbm/s
$\dot{m}$	evaporation rate of water, lbm/h
$N$	number of airfoils, dimensionless
$Nu$	Nusselt number, dimensionless
$n_{melt}$	melt fraction, dimensionless
$P$	pressure, psia
$P_a$	partial pressure of air, psia
$P_w$	partial pressure of water vapor, psia
$\bar{P}$	power, hp
$Pr$	Prandtl number, dimensionless
$Q$	volumetric flow, ft <sup>3</sup> /min
$\dot{q}$	heat flux, Btu/(s·in <sup>2</sup> )
$\mathcal{R}$	universal gas constant, 1545.5 ft·lbf/(lbm·mol·R)
$R$	Gas constant, ft·lbf/(lbm·R)
$r$	blade radius, in.
$r_H$	blade hub radius, in.
$r_T$	blade tip radius, in.

$r_c$	arc radius of blade at midchord, in.
Sc	Schmidt number, dimensionless
Sh	Sherwood number, dimensionless
$S_h$	specific humidity
$T$	temperature, R
$T_M$	averaged temperature of control volume, R
$t$	time, s
$U$	rotor peripheral velocity, ft/s
$V$	velocity, ft/s
$\tilde{V}$	control volume
$W$	relative velocity
$W_a$	flow rate of air, lbm/s
$W_w$	flow rate of water vapor, lbm/s
$w$	mass fraction, dimensionless
$X$	distance from blade leading edge to circular arc center coordinates, in.
$x$	distance, in.
$Y$	distance from blade trailing edge to circular arc center coordinates, in.
$\Phi$	rotor flow coefficient, dimensionless
$\alpha$	absolute flow angle, deg
$\beta$	edge angles, deg
$\gamma$	ratio of specific heats
$\gamma_{\text{wet}}$	ratio of specific heats of air and water mix, dimensionless
$\delta$	slip factor, dimensionless
$\eta$	rotor total-to-total adiabatic efficiency, dimensionless
$\theta$	blade arc angle, deg
$\lambda$	aerodynamic blockages, dimensionless
$\mu$	viscosity of air, absolute, lbm/(ft·s)
$\rho$	density, lbm/ft <sup>3</sup>
$\sigma$	blade tip solidity, dimensionless
$\tau_T$	thermal response time, s
$\phi$	relative humidity, dimensionless
$\chi$	mole fraction, dimensionless
$\omega$	loss coefficient, dimensionless

## Subscripts

1	leading edge rotor
2	trailing edge rotor (gap inlet)
3	leading edge stator (gap outlet)
4	trailing edge stator
$a$	air
$B$	blade metal
convection	
$d$	ice particle diameter

evaporation  
exit  
*F* flow  
*f* freezing  
*H* hub  
*i* ice  
inlet  
*M* meridional  
melt ice melt  
*o* initial  
*R* rotor  
*S* static  
*s* water vapor at surface or sublimation  
*sat* saturated  
*T* tip or thermal  
*t* total  
*U* tangential  
*V* stator vane  
*v* evaporation  
*WB* wet-bulb  
*WBS* wet-bulb static  
*w* water vapor  
wet air and water mixture  
 $\infty$  free-stream

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