

# THE IMPACT OF PACKAGING ON SOFT MAGNETIC CORE PERFORMANCE

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# Why Packaging?

TRL 9

•Actual system "flight proven" through successful mission operations

TRL 8

•Actual system completed and "flight qualified" through test and demonstration (ground or flight)

TRL 7

•System prototype demonstration in a flight environment

TRL 6

•System/subsystem model or prototype demonstration in a relevant environment (ground or flight)

TRL 5

•Component and/or breadboard validation in relevant environment

TRL 4

•Component and/or breadboard validation in laboratory environment

TRL 3

•Analytical and experimental critical function and/or characteristic proof-of-concept

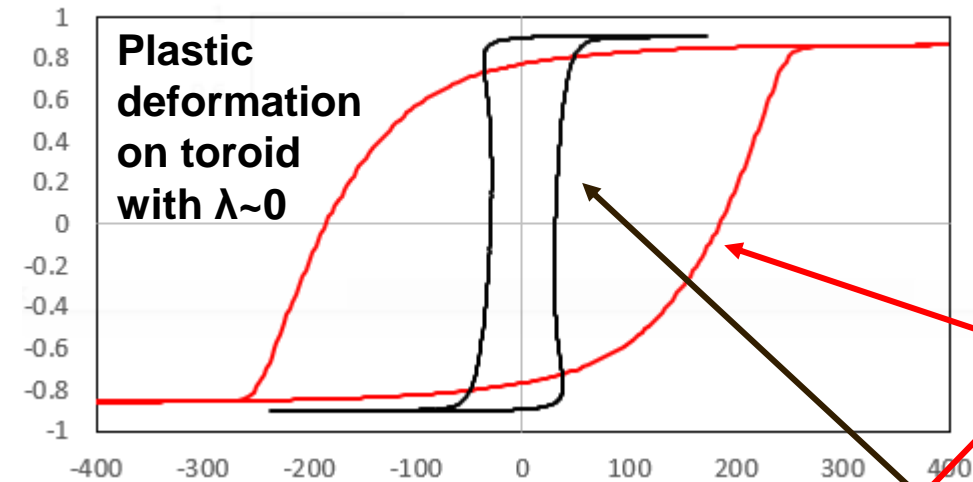
TRL 2

•Technology concept and/or application formulated

TRL 1

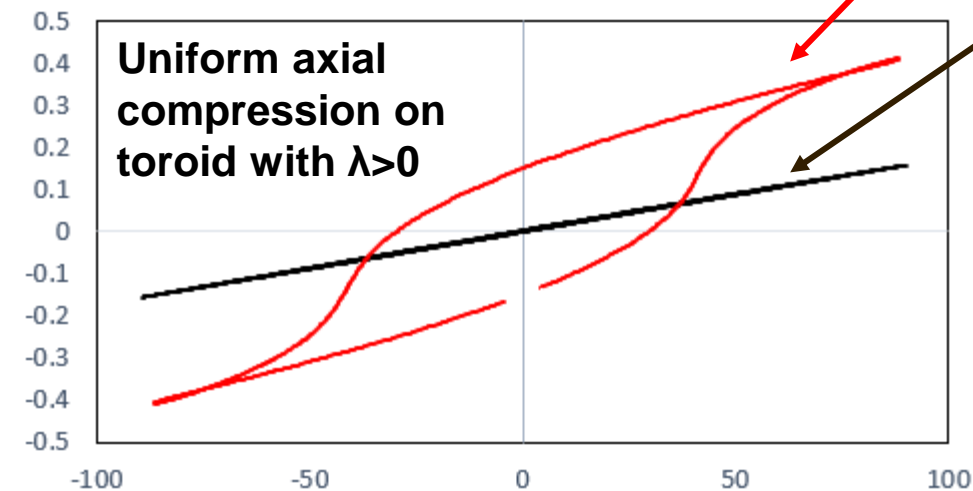
•Basic principles observed and reported

Square Permalloy 80 1600Hz B(T) vs H(A/m)



Applied stress

FeCo-based MANC 400Hz B(T) vs H(A/m)



No stress

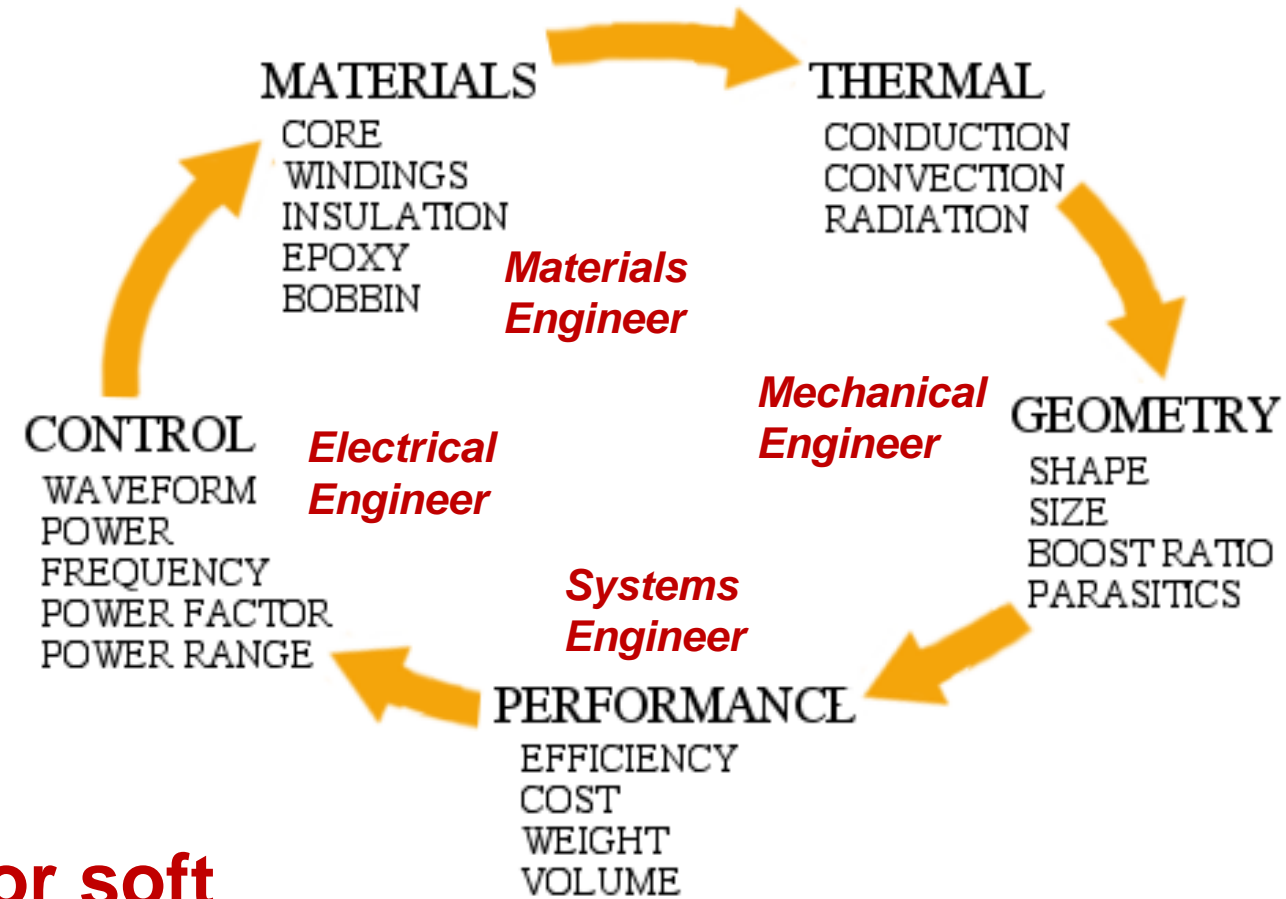
# Overview

**Application:** Magnetics for MW scale power conversion with high power density, usually kHz→MHz

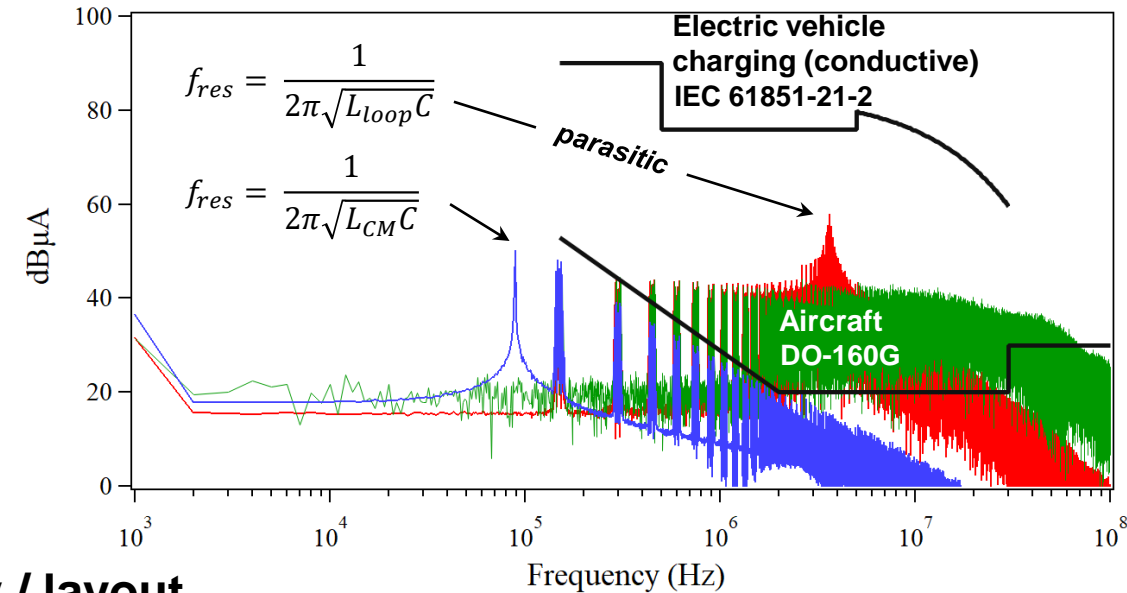
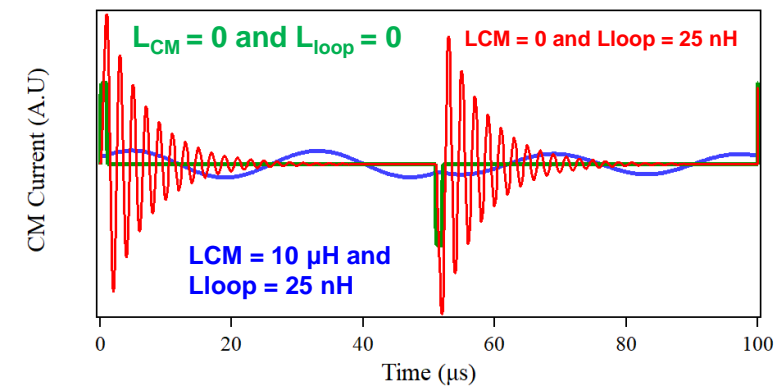
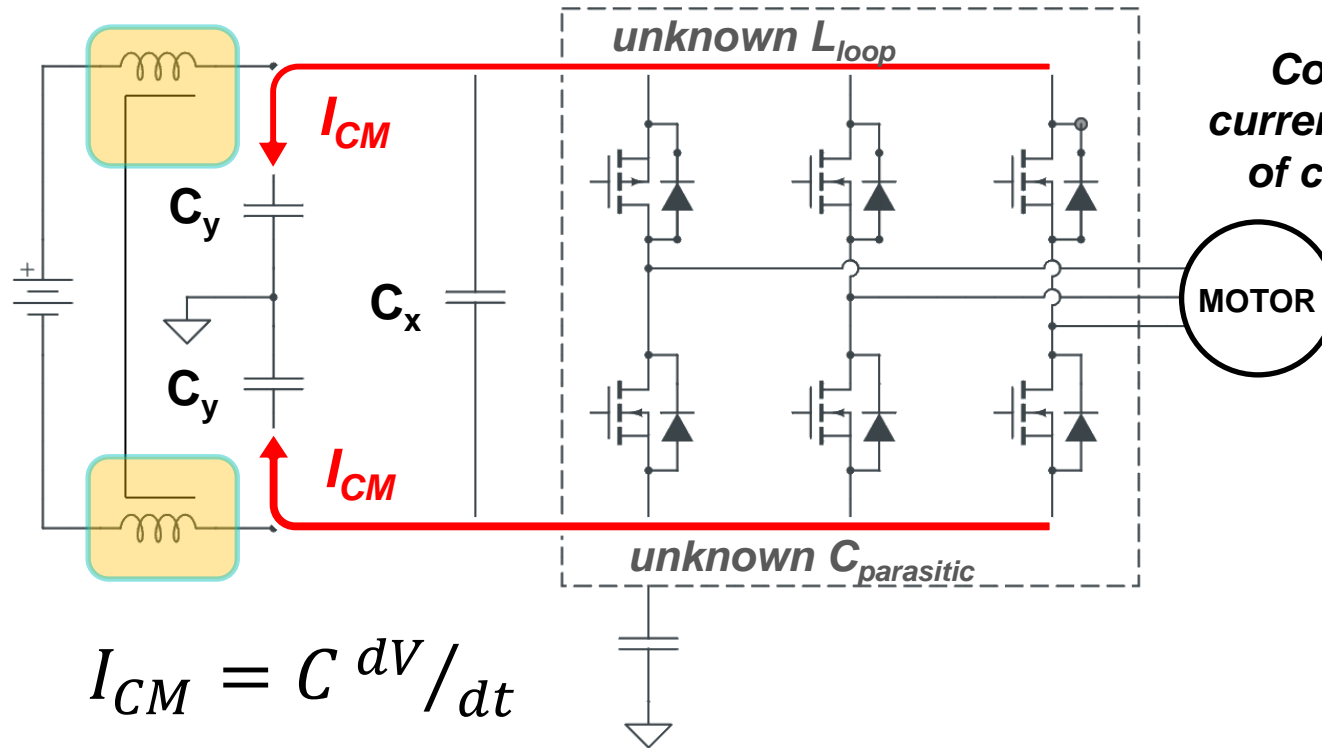
**Challenges:**

1. Operation closer to margin limits and/or integration of new materials and components at the subsystem level
2. How to simulate the design space efficiently to enable complex systems?

**Goal:** Develop design guidelines for soft magnetic nanocomposite core packaging



# Common Mode Noise in Simple Electric Vehicle



- Parasitic impedances related to components and geometry / layout
- $C_y$  is limited by safety to  $\sim 1\mu F$  for  $<1kV$
- Fast  $dV/dt$  in WBG and UHBG semiconductors creates large CM currents in ground planes and chassis

Conducted EMI standards control noise  $f > 150kHz$  with parasitic resonances common in the 1-10 MHz range  
Performance, safety, and reliability related to topology, layout, controls, and materials



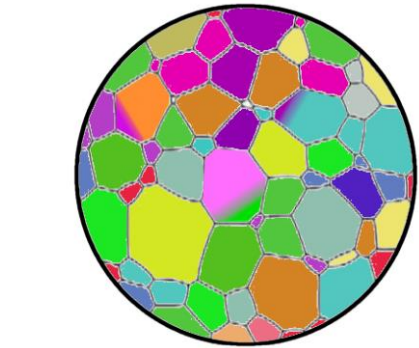
# What do we need from packaging?

- **Decouple winding stress from the core material**
- **Provide mechanical mounting to withstand vibration and shock**
- **Provide adequate thermal path**
- **Provide adequate electrical insulation**
- **Fit into the surrounding components**

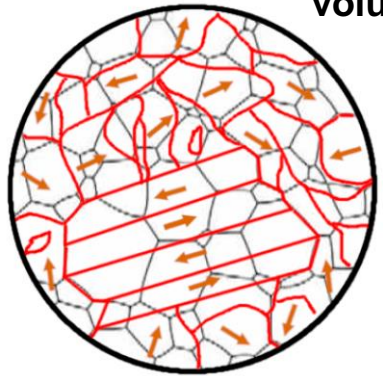


# Hysteresis: Magnetic and Mechanical

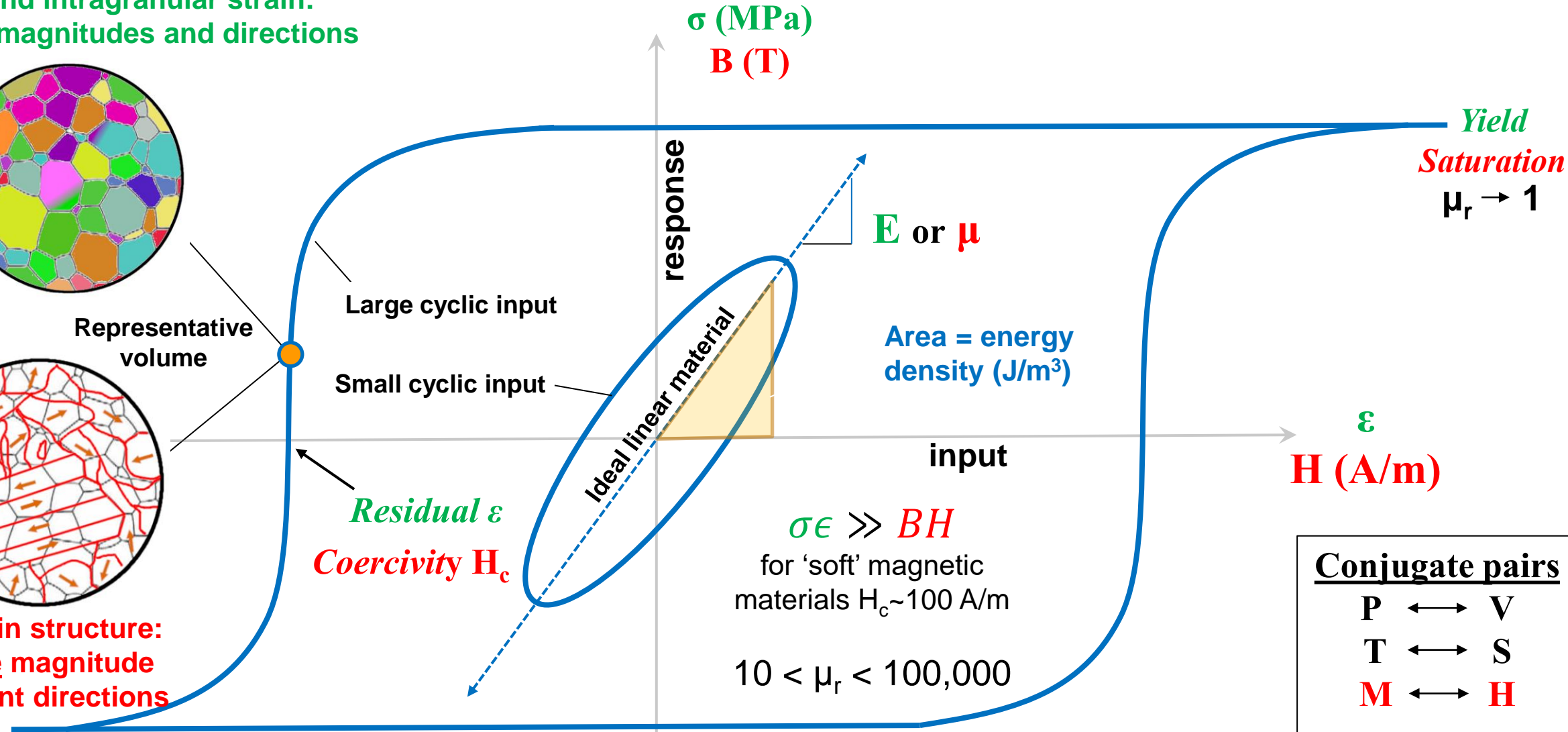
Inter and intragranular strain:  
Different magnitudes and directions



Representative volume



Domain structure:  
Same magnitude  
different directions



## Conjugate pairs

P	↔	V
T	↔	S
M	↔	H
σ	↔	ε

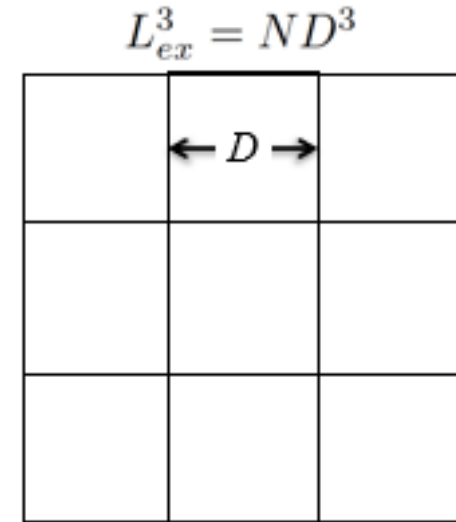


## Single phase Random Anisotropy Model

$$\begin{bmatrix} \varepsilon \\ P \\ B \\ \Delta S \end{bmatrix} = \begin{bmatrix} C_{\varepsilon}^{\sigma} & C_{\varepsilon}^E & C_{\varepsilon}^H & C_{\varepsilon}^{\Delta T} \\ C_P^{\sigma} & C_P^E & C_P^H & C_P^{\Delta T} \\ C_B^{\sigma} & C_B^E & C_B^H & C_B^{\Delta T} \\ C_{\Delta S}^{\sigma} & C_{\Delta S}^E & C_{\Delta S}^H & C_{\Delta S}^{\Delta T} \end{bmatrix} \begin{bmatrix} \sigma \\ E \\ H \\ \Delta T \end{bmatrix}$$

*magnetostriction*

*piezomagnetic*



$$\langle K \rangle = \frac{K_1^4 D^6}{\psi_0^6 A^3}$$

Characteristic length  
scale of applied stress  
is usually 'global'  
compared to grain size

- Coupling terms  $B = \mu H$  and  $\varepsilon = E\sigma$  measurable for static conditions
- Dynamic conditions can be described using complex  $\mu^*$  or  $E^*$

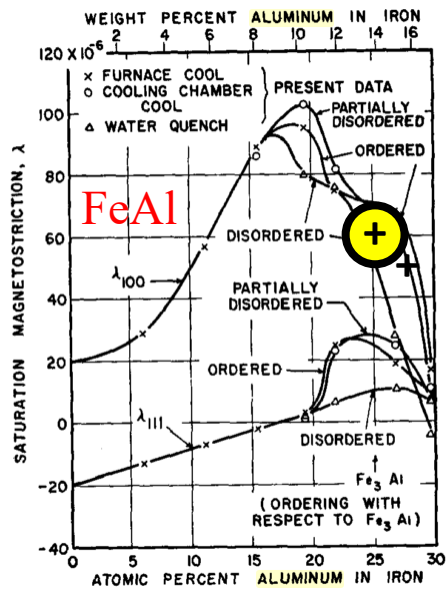
What about dynamic conditions for  $B = C_B^{\sigma*} \sigma + \mu^* H \dots$

Often, we want  $C_B^{\sigma*} \rightarrow 0$

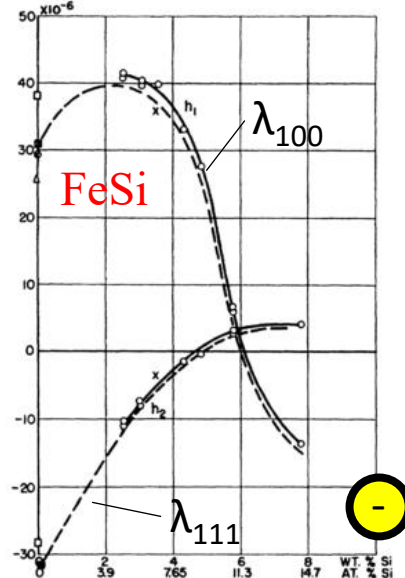
Opposing magnetostriction:  $\lambda_1 v_1 + \lambda_2 v_2 \sim 0$

Sendust ( $\text{Fe}_{74}\text{Al}_{11}\text{Si}_{15}$ )  $\sim v_{\text{Fe}_3\text{Al}} + v_{\text{Fe}_3\text{Si}}$

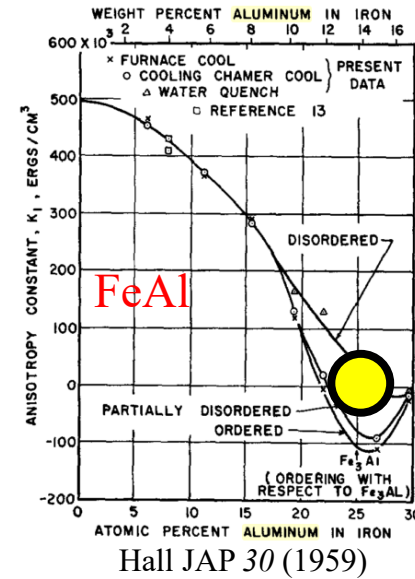
$\lambda_s > 0$        $\lambda_s < 0$



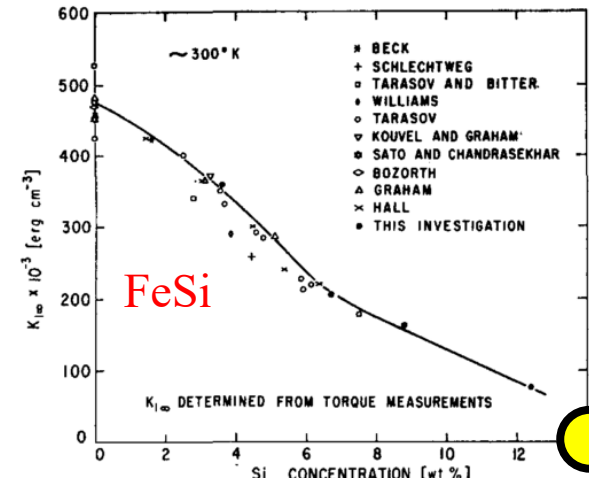
Hall, JAP 30 (1959)



Carr & Smoluchowski, *Phys Rev* 83 (1951)



Hall JAP 30 (1959)



Arajs, et al JAP 32 (1961).

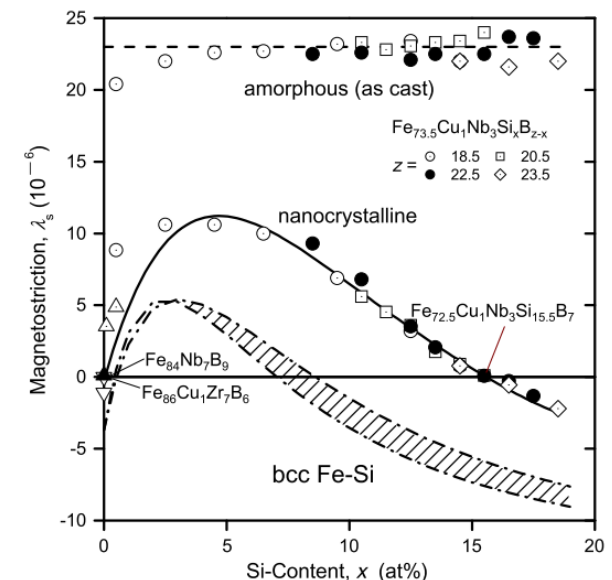
... both have small  $K_1$

Same mechanism used in Finemet...

$\lambda_s > 0$        $\lambda_s < 0$

$v_{\text{amorphous}} + v_{\text{Fe}_3\text{Si}}$

Here, average  $\langle K \rangle \sim 0$   
by the random  
anisotropy principle  
and Si partitions to the  
crystal



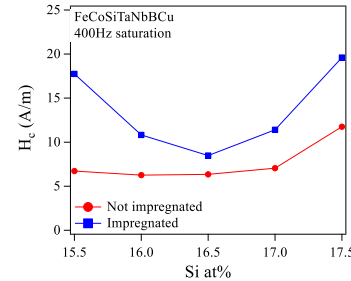
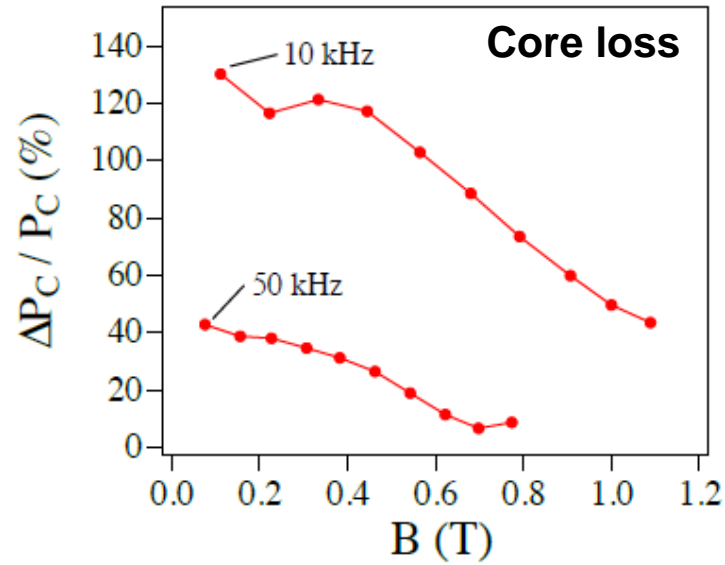
Herzer, *Acta Mat* 61 (2013)

We can reduce magneto-elastic coupling either with zero magnetostriction material or by mixing positive and negative magnetostrictive phases in a composite



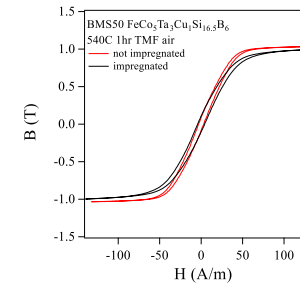
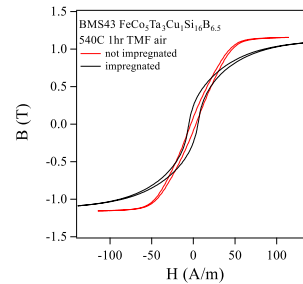
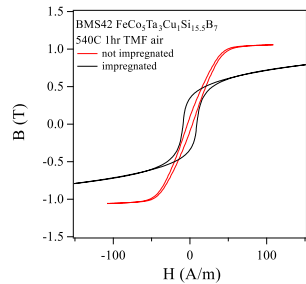
# Epoxy Impregnation Stress

$$\lambda_{\text{nanocomposite}} = v_{\text{crystal}} \lambda_{\text{crystal}} + (1 - v_{\text{crystal}}) \lambda_{\text{matrix}}$$



**Material data sheet core losses do not include packaging stress.**

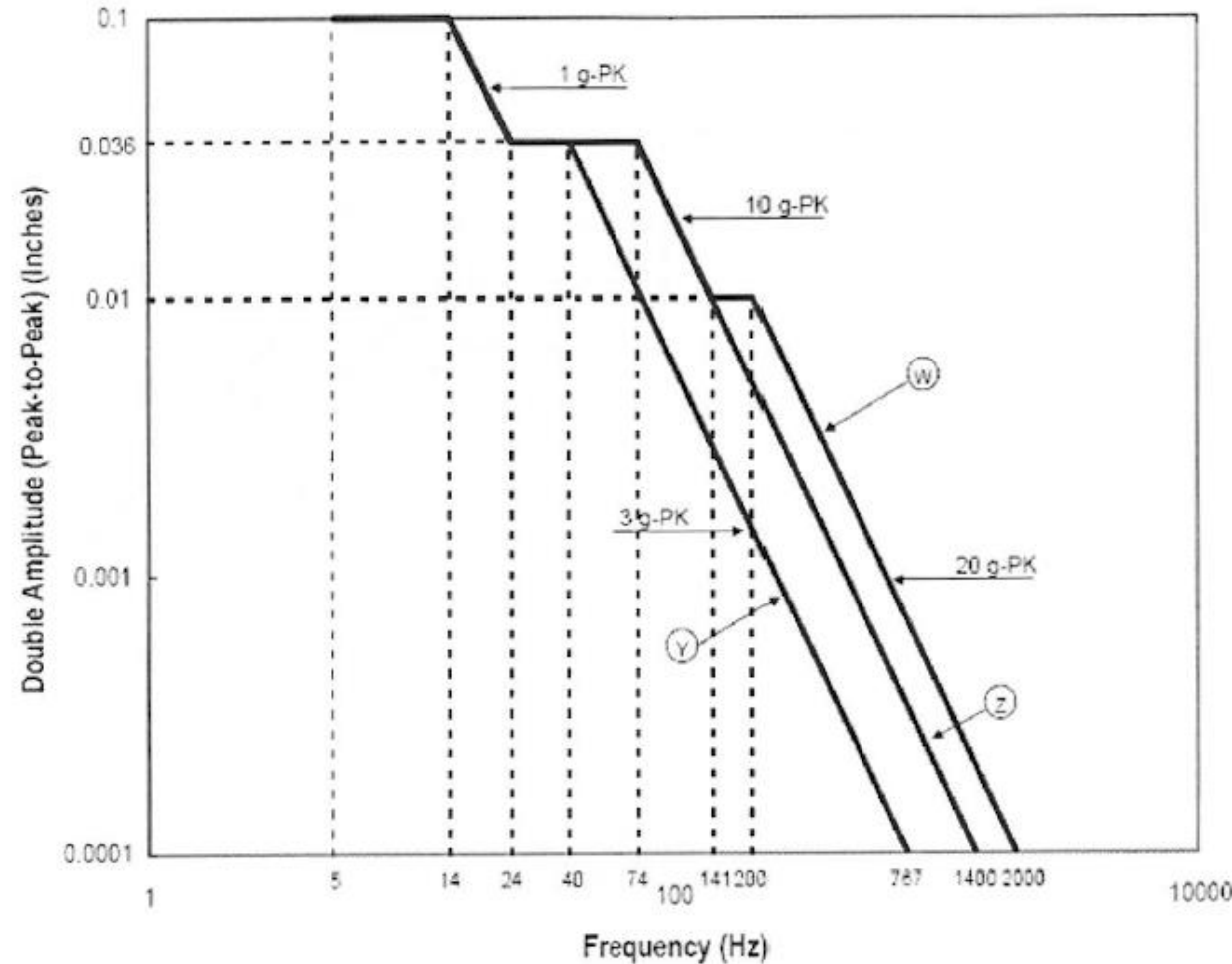
**In FeSi-based nanocomposites, we can balance volume fraction ( $v$ ) with magnetostriction ( $\lambda$ ) of crystal and matrix phases.**



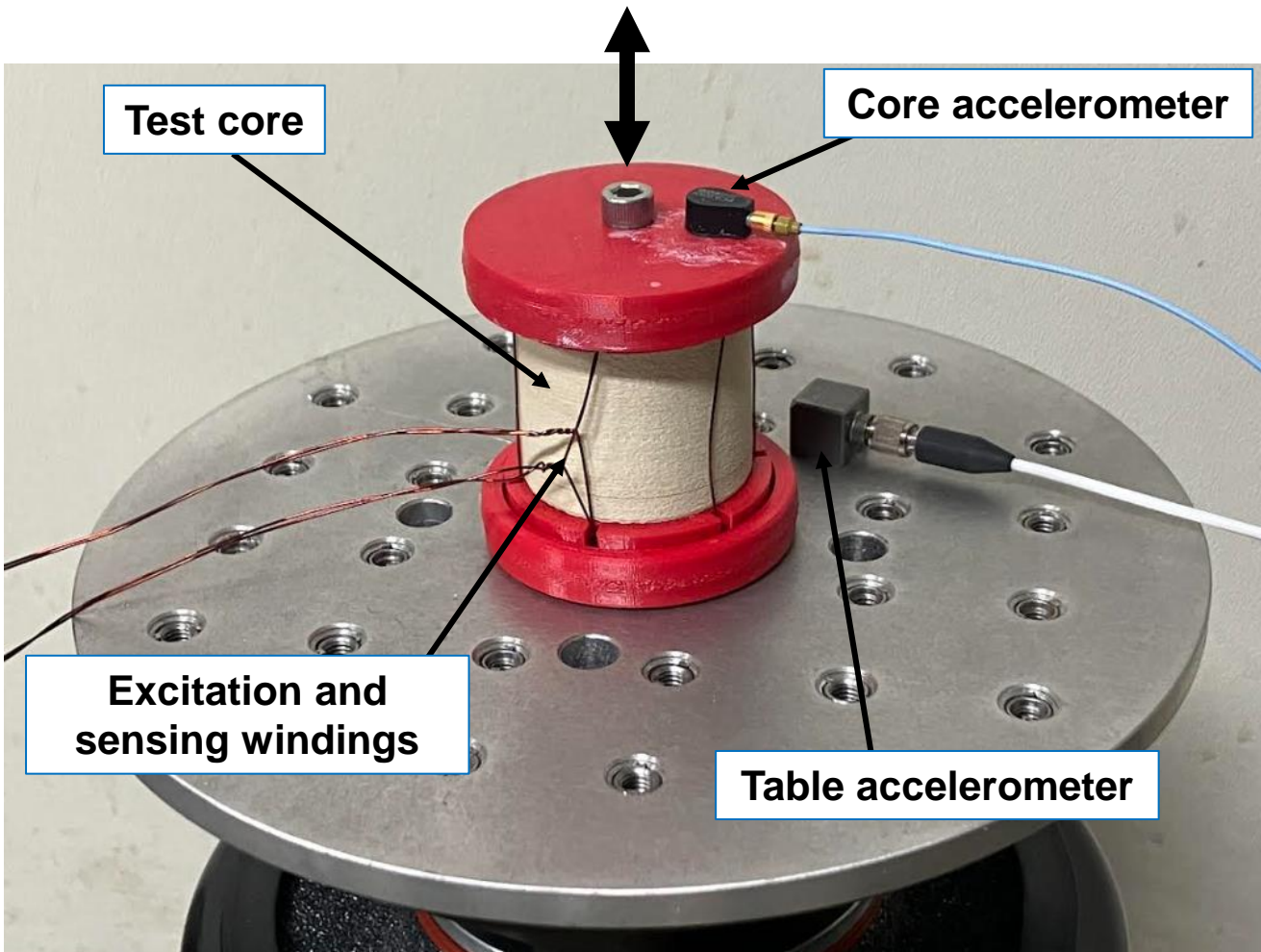
**Increasing Si**

# Mechanical and Thermal Requirements

- Thermal: Typically, <150C with heat sinks as required.
- Mechanical: 3 axis vibration 20g through 2kHz (DO-160G Section 8)
  - Vibration + electrical function
  - Usually measured in components
- Packaging Options: Casings, potting, windings, mounts
- Problems within components and problems in neighboring components...



# Vibration Stresses



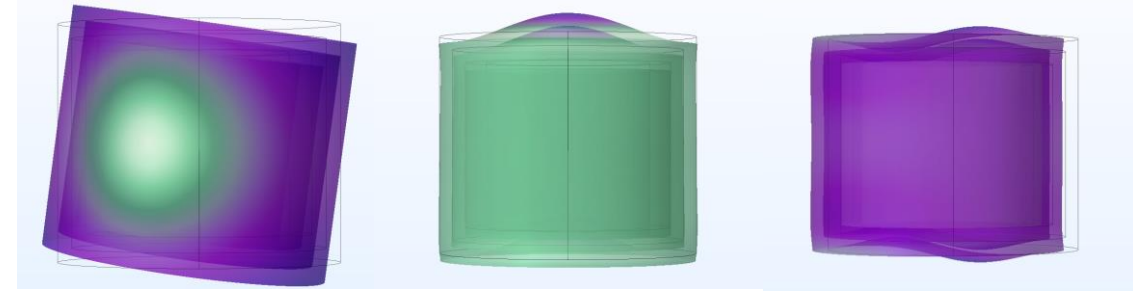
Vibration stresses are small, but resonant modes *could* produce high localized core stress

## FEA resonant modes: Box + potting + core

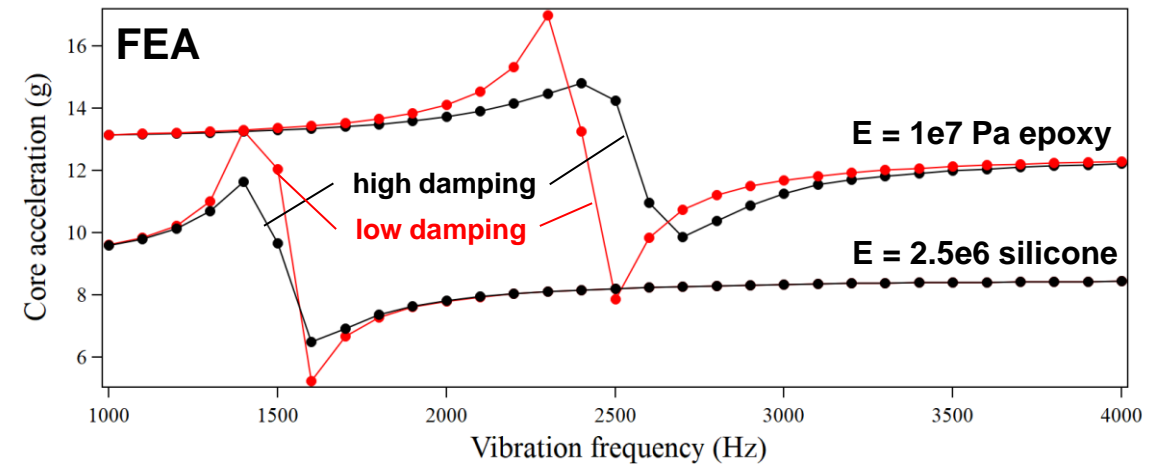
$f_1 = 1384 \text{ Hz}$

$f_2 = 2410 \text{ Hz}$

$f_3 = 2832 \text{ Hz}$

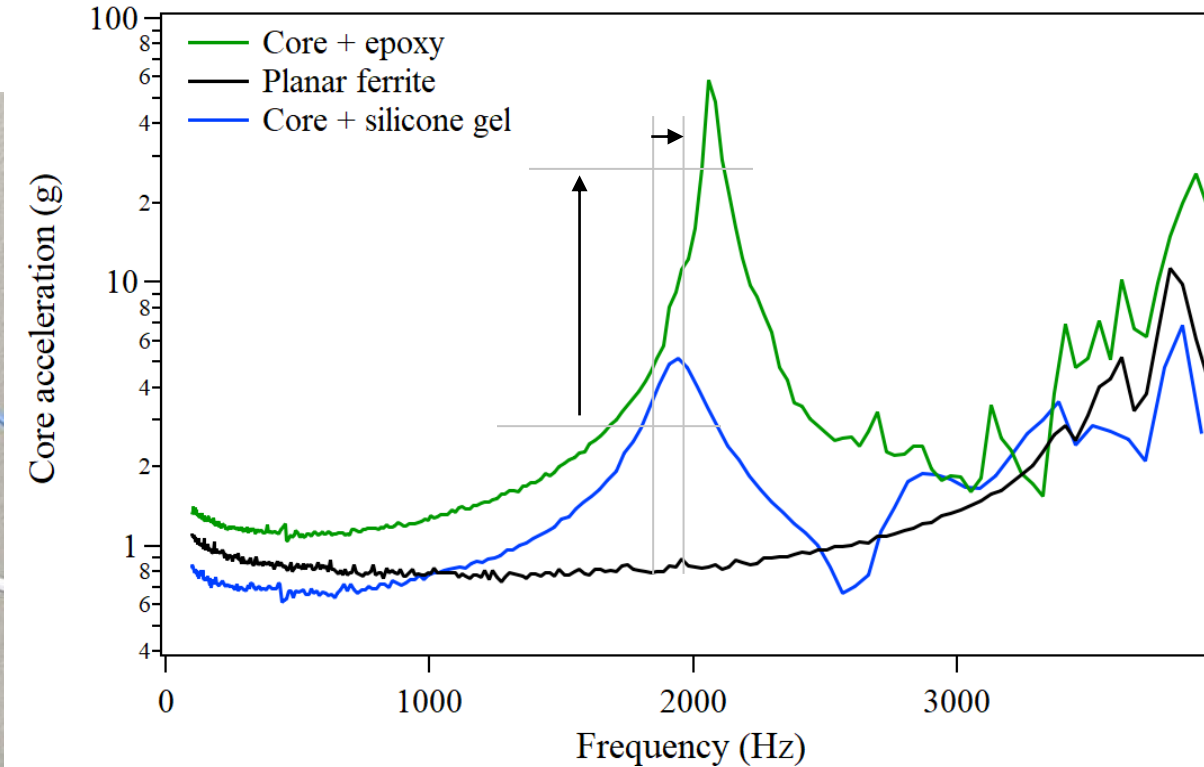
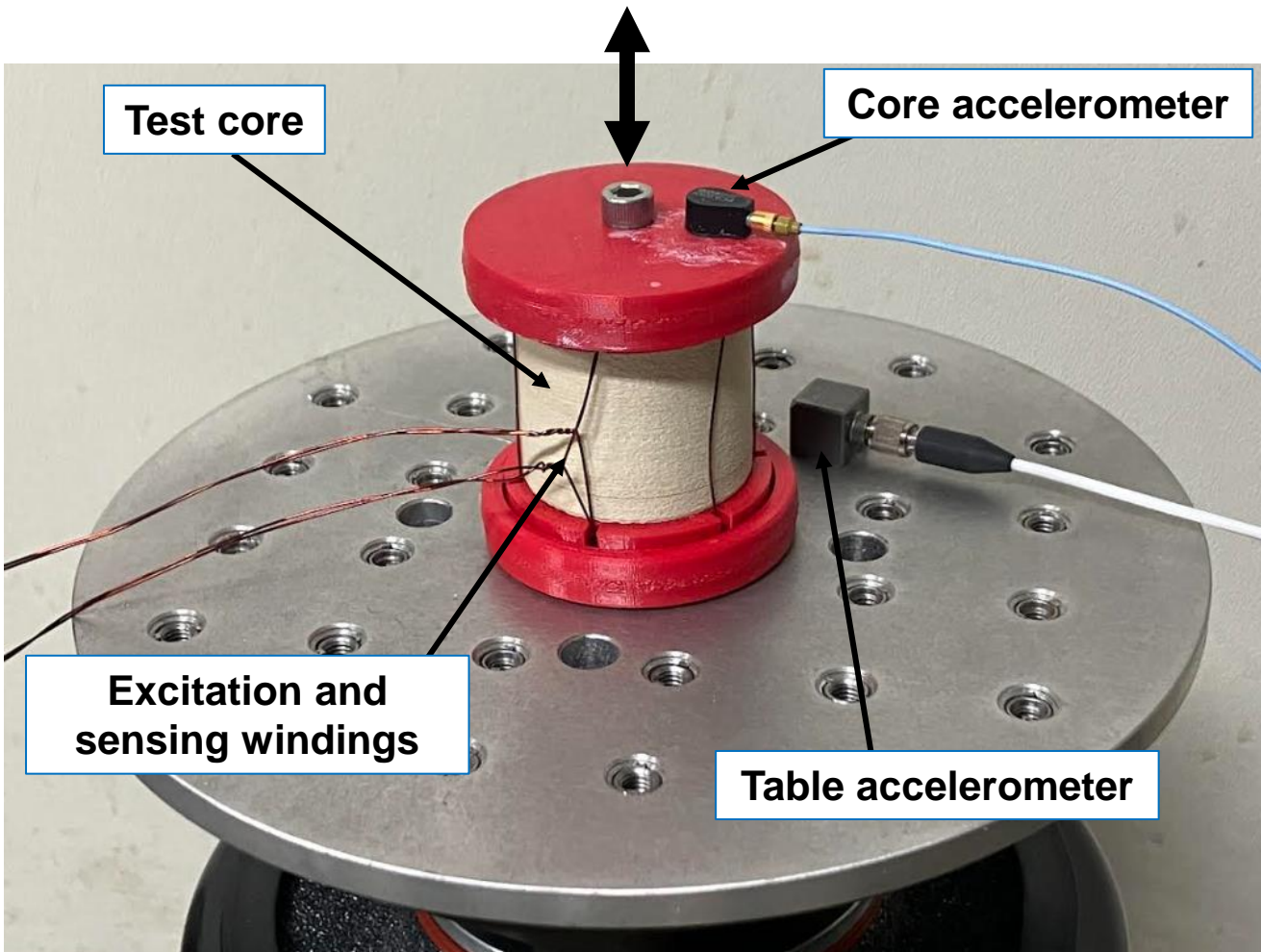


$F = ma \sim 100\text{N}$  for a 0.1kg core in 100g  
 $\sigma = 0.34 \text{ MPa}$  over the bottom surface of the toroid





# Vibration Stresses



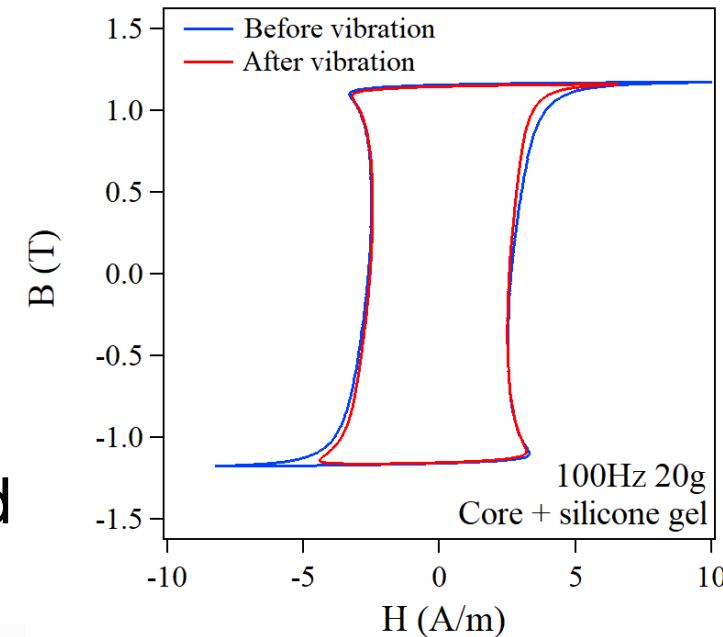
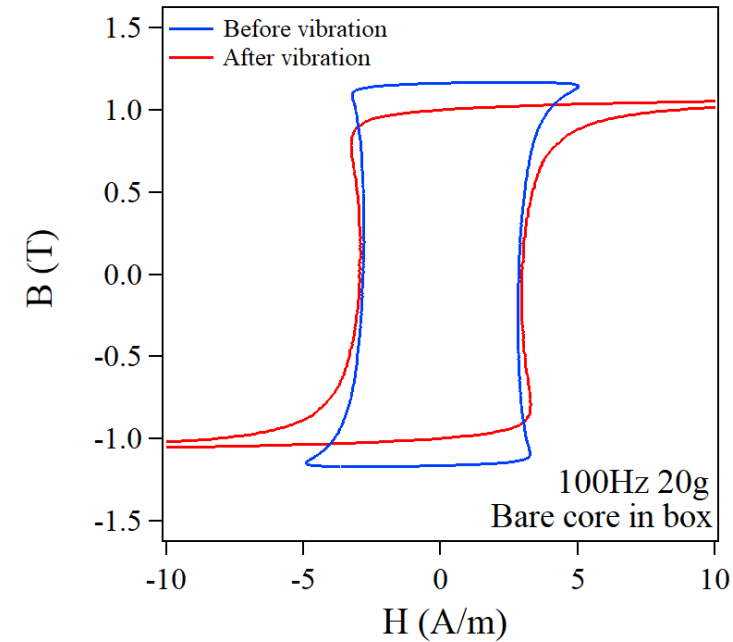
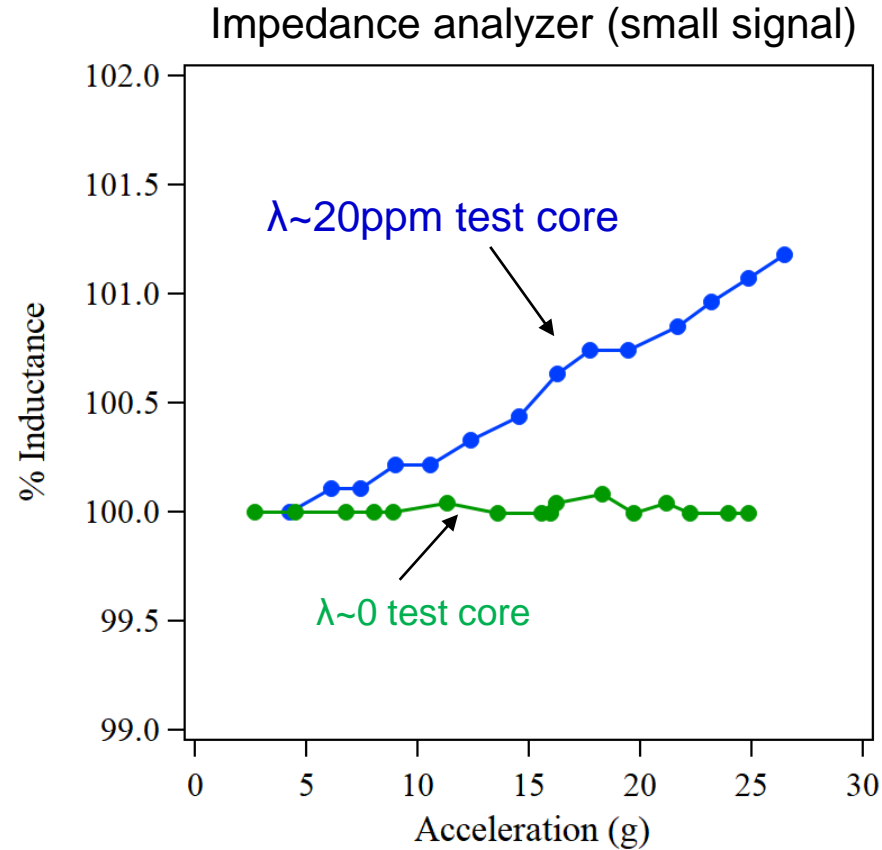
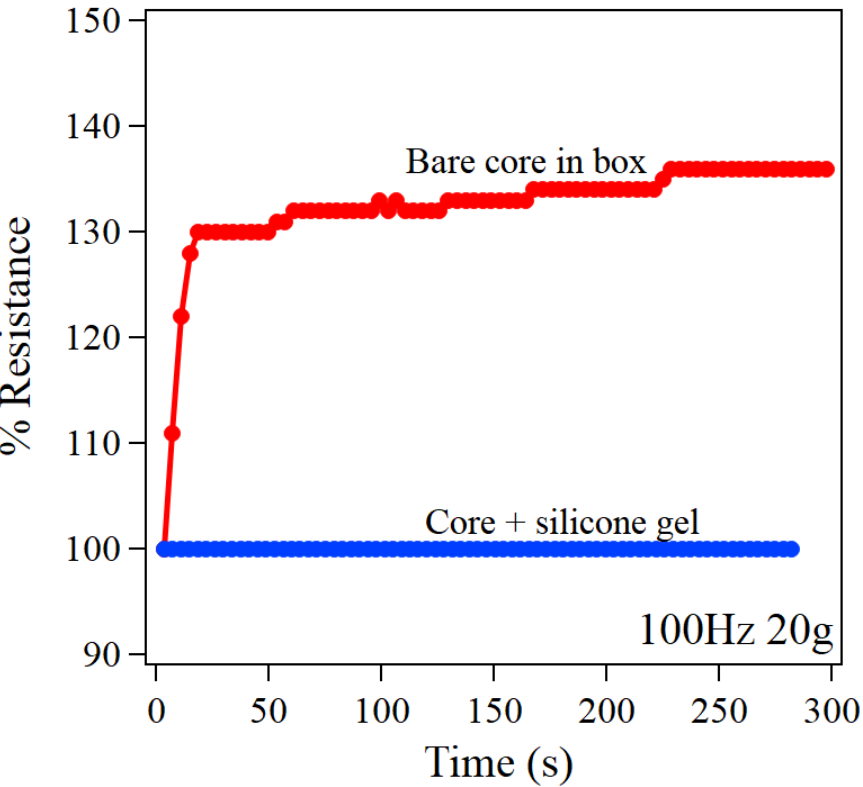
**Silicone potting produces lower resonant frequency and lower acceleration compared to epoxy potting.**

**Comparison with rigid mount:  
STANDEX ferrite inductor**



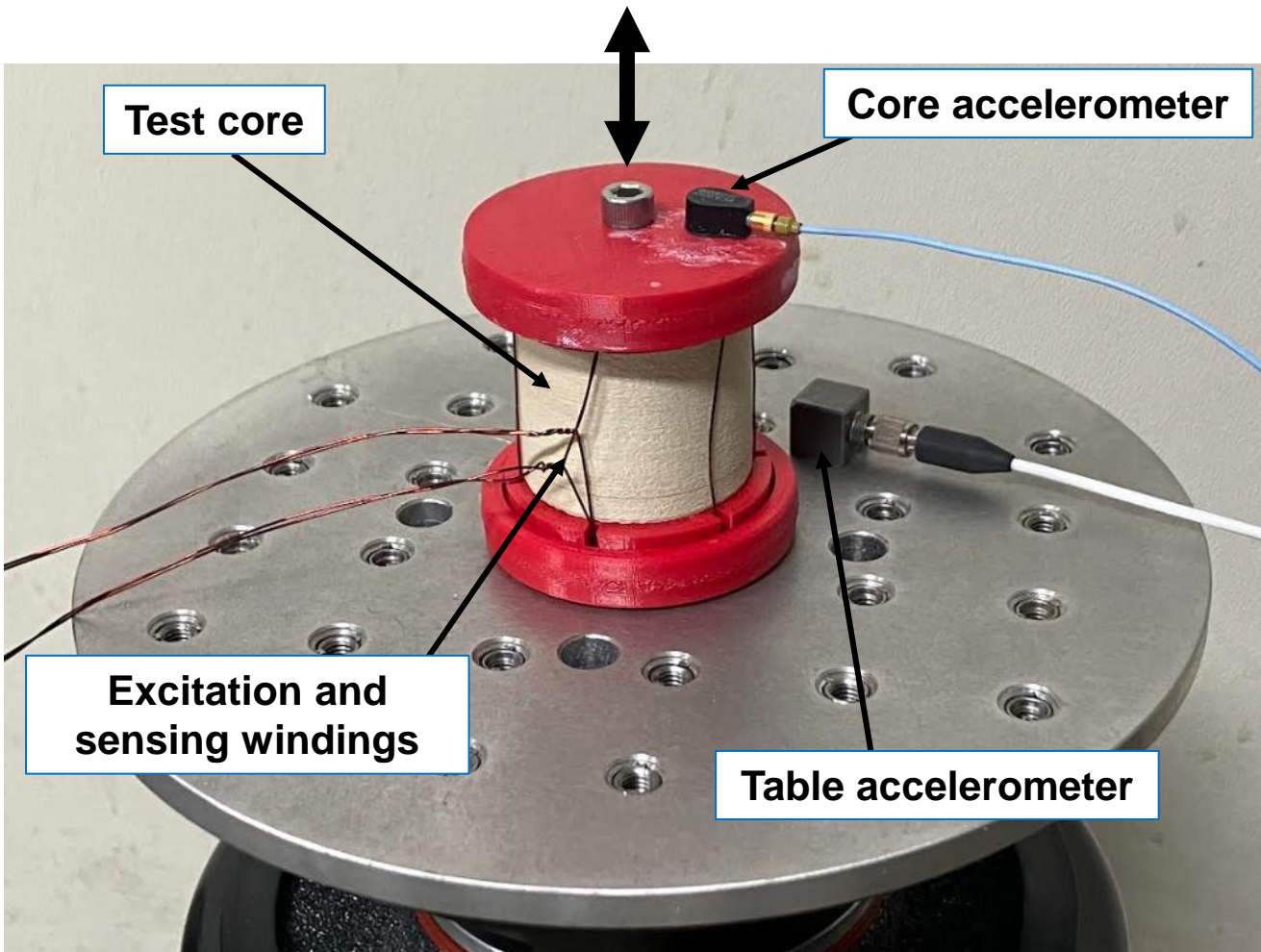


# Vibration Stresses

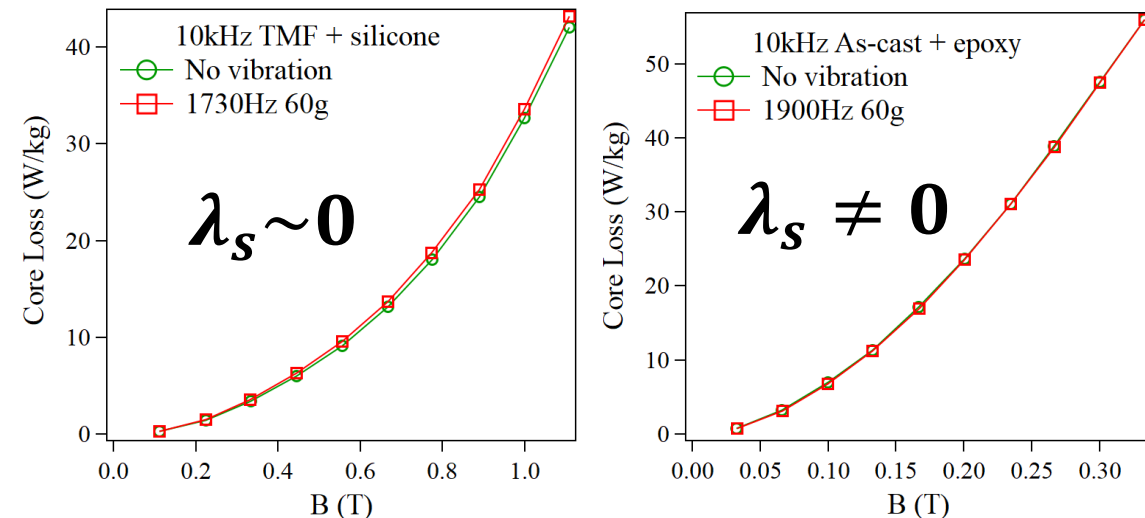
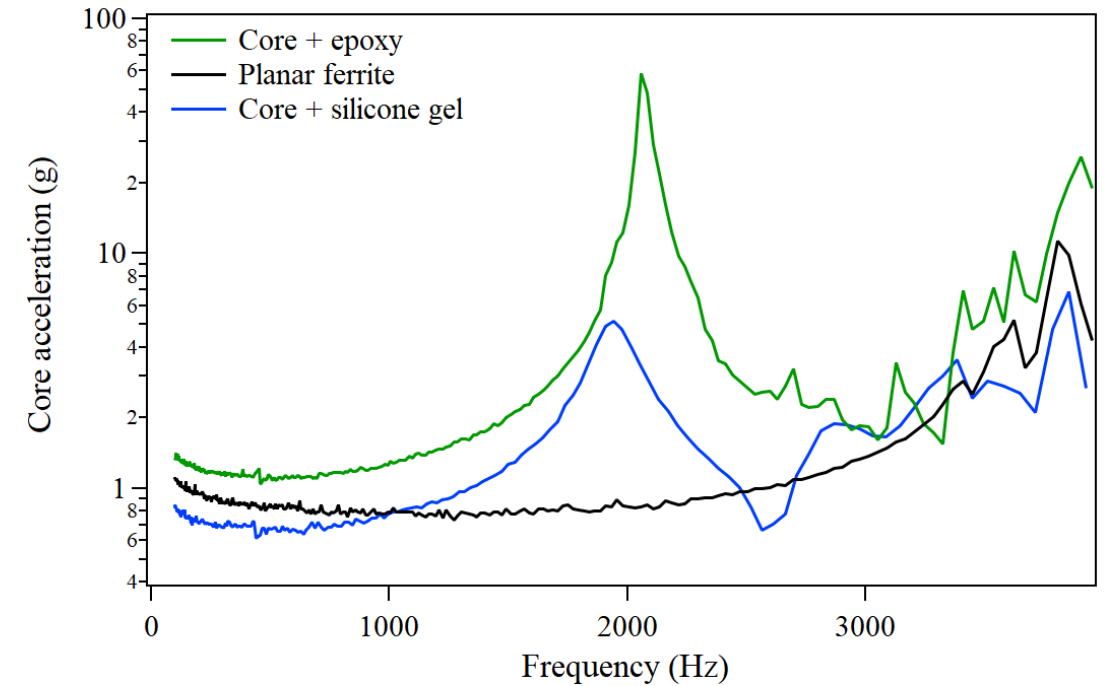


- Negligible change  $\sim 1\%$  in inductance from 20g 100Hz vibration in z-axis for gel potted cores
- Measured  $>10\%$  inductance change for epoxy impregnated cores vibrated in x-axis

# Vibration Stresses

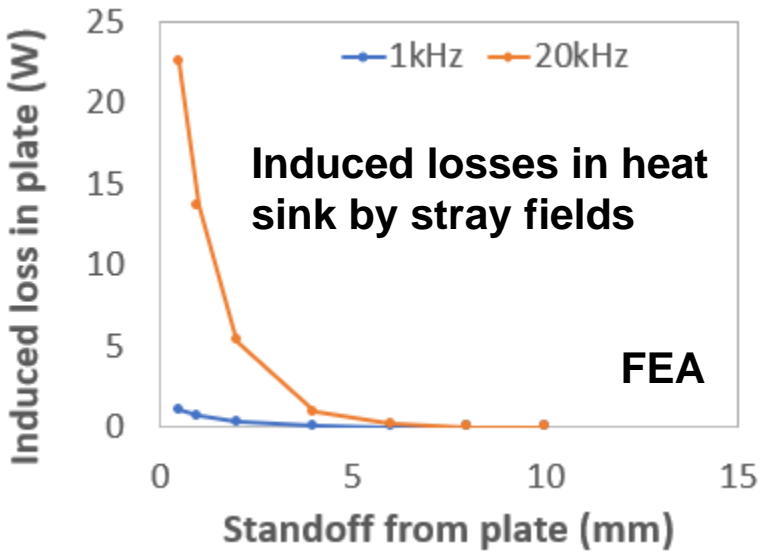
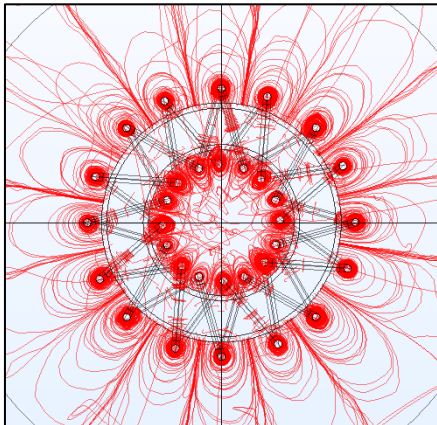
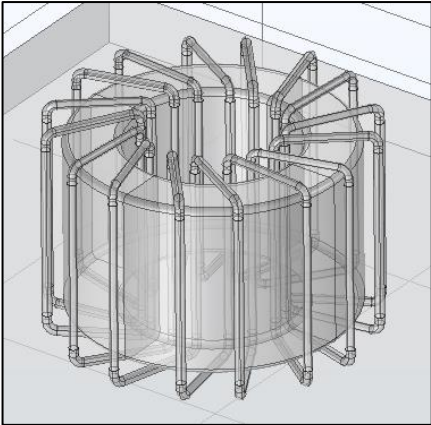
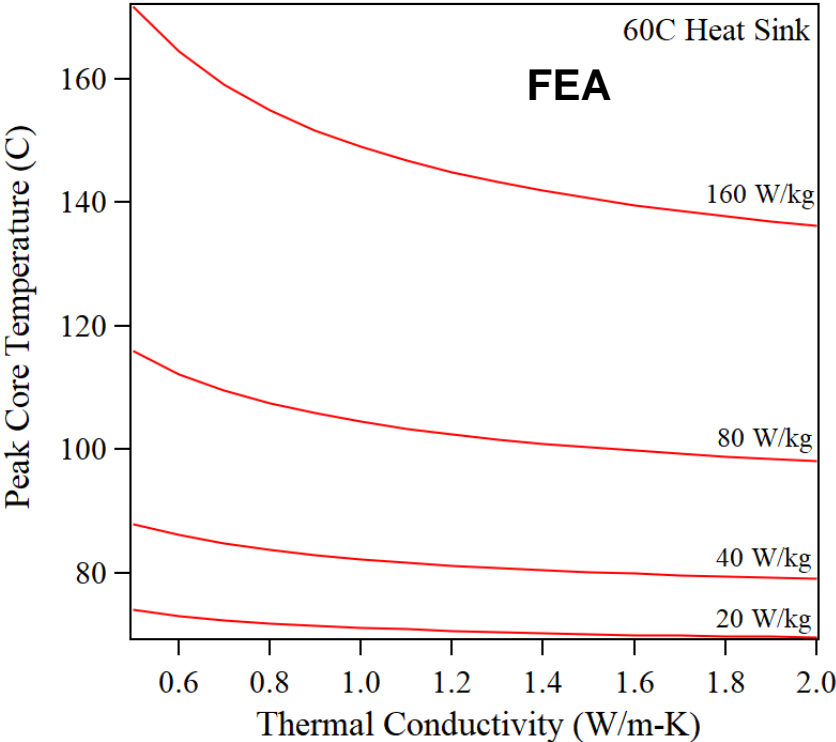
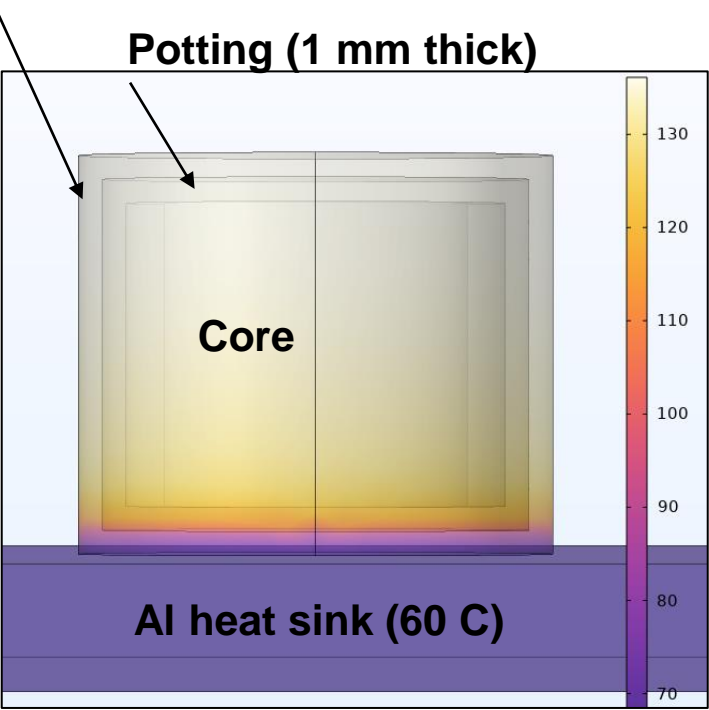


**Negligible change in measured losses at peak resonant frequency... stress is small**



# Heat Transfer and Electrical Conductivity

Core box (31x25x25 square toroid)



For small potting dimensions, thermal conductivity of potting is small compared to core losses. High thermal (electrical) conductivity materials near windings may produce significant loss.



# X-57 EMI Filters



## Design requirements

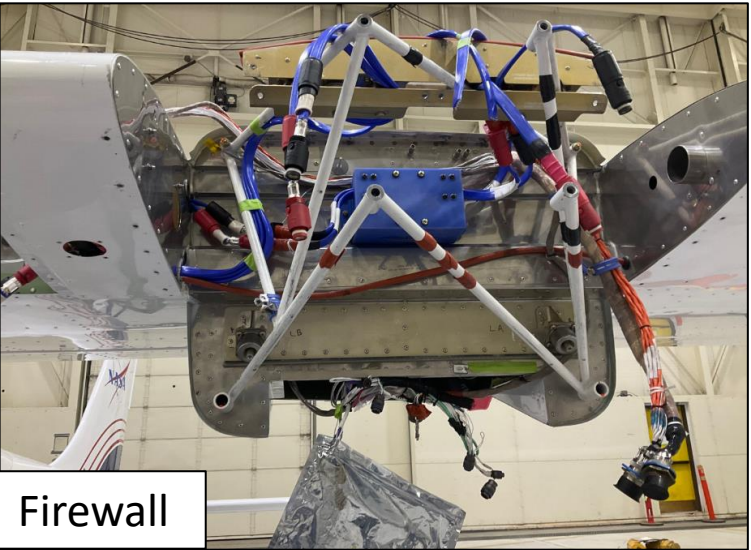
$L = 14\mu\text{H}$ ,  $I_c = 10\text{A}_p$ ,  $ID > 17\text{ mm}$

## Design comparison

	Ferrite (SOA)	Nanocomposite
Toroid	21x53x51 (mm)	21x40x25 (mm)
$B_m$ (T)	0.3	0.8
$\mu_r$	1600	5300
Mass (g)	454 ← 3X →	145

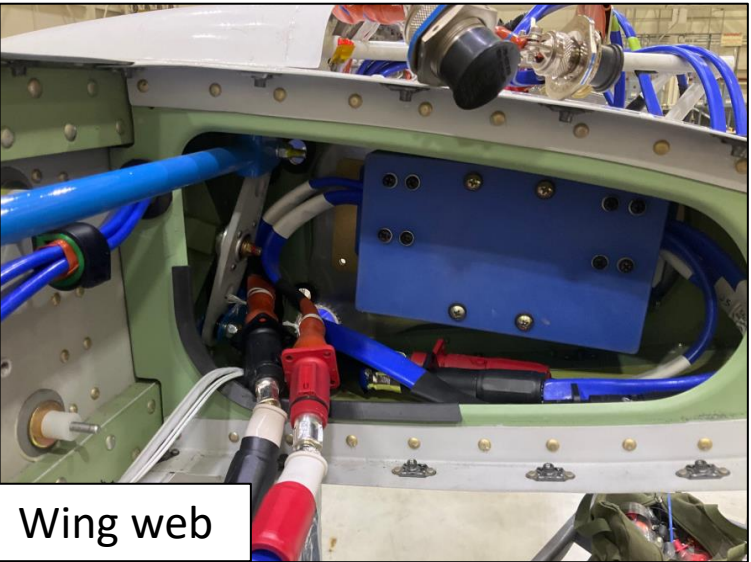
Reduction in size and mass – compared to SOA

Filter design passed vibration test at Armstrong Flight Research Center

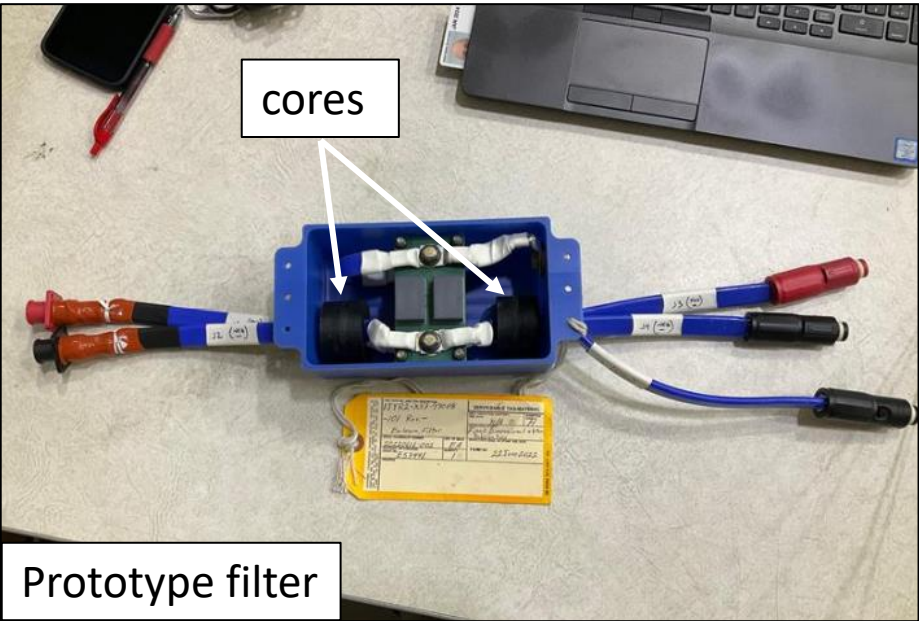


Firewall

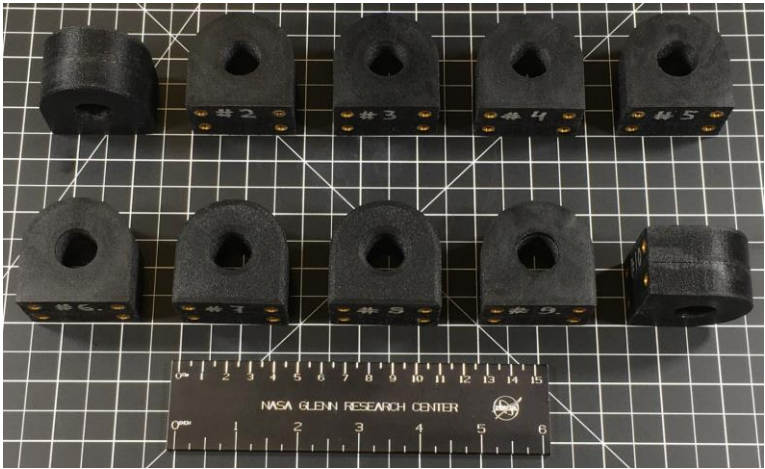
X-57 mounting locations



Wing web

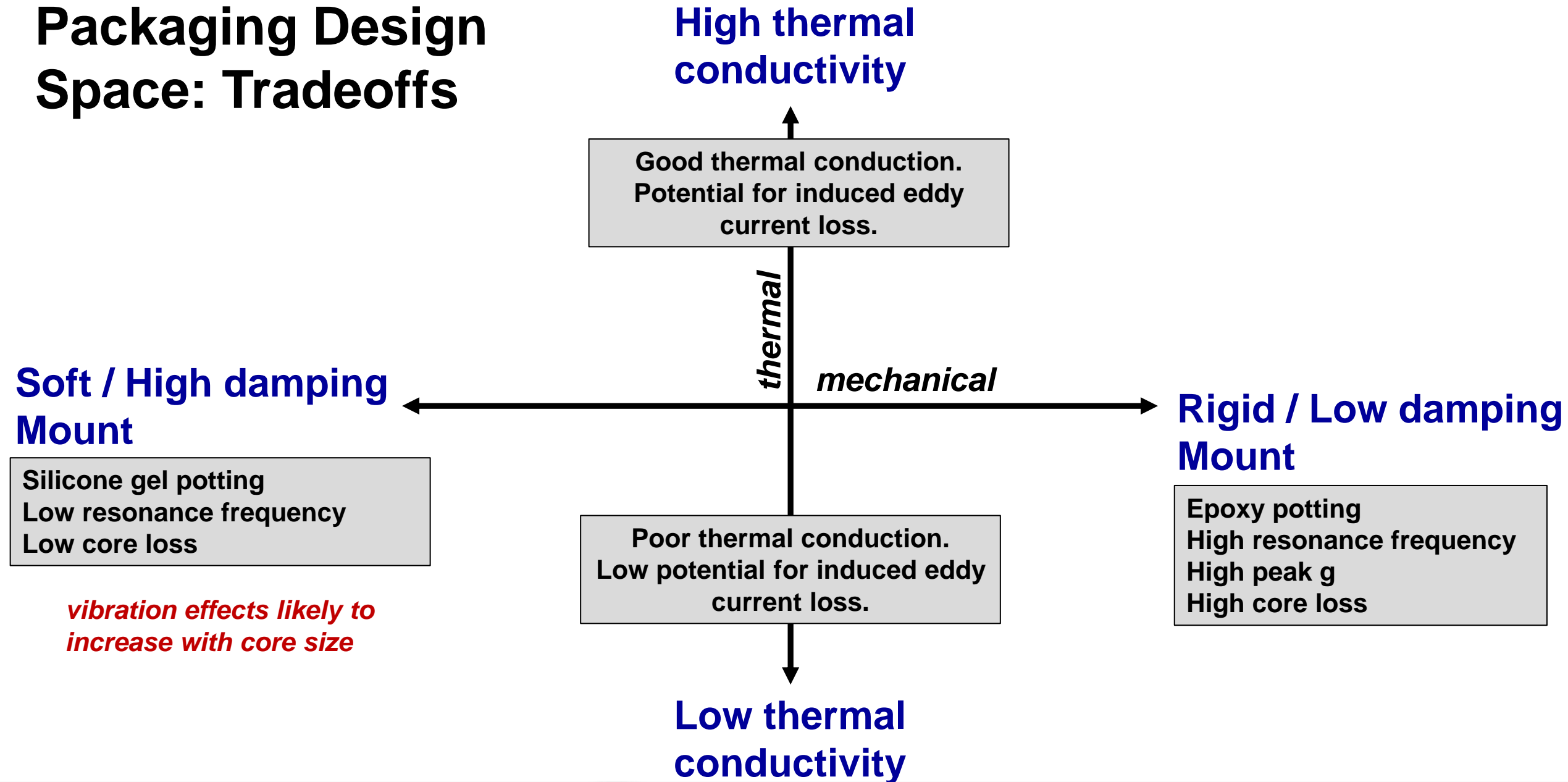


Prototype filter





# Packaging Design Space: Tradeoffs



# Conclusions

- Each design is different and should be validated
- Silicone gel can be an effective potting material for fragile MANCs under DO-160G vibrations
- Softer mounts can shift vibrations to lower frequency in neighboring components
- Design Tradeoffs:
  - high/low thermal and electrical conductivity
  - Soft/rigid mounting

Thank you to NASA AATT Electrified Aircraft Propulsion