### Testing and Analyses of Advanced Composite Tow-Steered Shells with Cutouts

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# **Talk Outline**

- Introduction
- Baseline shells w/o cutouts
  - Design and manufacturing
  - Testing and analyses
- Shells with small cutouts
  - Cutout description
  - Compression tests and finite element analyses
  - Correlation of results
    - Global, local
- Shells with large cutouts
  - Correlation of global results
- Summary and concluding remarks

# **Study Objectives**

- Assess structural performance of tow-steered composite shells with small cutouts
  - Shells with and w/o tow overlaps
  - Same nominal layup
- Test shells in quasi-static end compression
  - Prebuckling deflections and strains
  - Postbuckling deflections and strains
- Compare nonlinear structural analysis results with corresponding test data
  - Discrete locations on shell planform
  - Measured analog (LVDT, strain gage) and digital image correlation (DIC) deflections and strains
  - Shell postbuckling behavior

## What Are Advanced Composites?

- In conventional composite laminates, all fibers in a ply are straight and parallel with a fixed orientation; structural tailoring is achieved by varying the numbers of plies and their relative orientations
- In advanced composites (a.k.a. *tow-steered* or *variable stiffness*) the fiber orientation within each ply can vary continuously over the structure's planform
- These configurations provide new opportunities for optimized design by tailoring load paths, thermal and mechanical properties, and damage tolerance
- Fiber placement systems that can precisely and accurately steer composite tows during manufacture are enabling technology for cost-effective fabrication of highly tailored structures

Baseline Shells w/o Cutouts Design, Manufacturing, Testing, and Analyses

## **Tow-Steered Shell Concept**



Principal fiber path defined as a constant-radius **circular arc** 

- start angle  $\Theta_0$  on crown/keel
- end angle  $\Theta_1$  on sides

Arc width = mandrel circumf./4

Design laminates to replicate "I-beam" bending response

- shell crown/keel carry axial compression/tension
- sides carry shear loads
- => Circumf. angle variation



**Fiber placement system** has minimum steering radius => manufacturing constraint

## **Design for Manufacturing**



## **Tow-Steered Shell Fabrication**





# **Shell Compression Test Set-up**

Nominal shell dimensions

- Overall length = 35.00 inches
- Inner diameter = 16.290 inches

Measured shell weights

- Shell with overlaps = 5.23 lbs
- Shell without overlaps = 4.13 lbs





Epoxy potting compound cast on shell ends to prevent brooming

4 displacement transducers measure relative platen motion

- Average = end shortening
- Diff's = transv. bending

56 back-to-back strain gage pairs bonded to shell surfaces

Digital image correlation systems also used to image shells during tests

## **Shell with Overlaps Compression Test**





## Shell with Overlaps Nonlinear FEA



### Shell w/o Overlaps Compression Test





## Shell w/o Overlaps Nonlinear FEA



## Six Shell Configurations Tested and Analyzed

## Includes Cutouts?

Shell with Overlaps	No	<u>Small</u>	Large
Shell w/o Overlaps	No	Small	Large
	Baseline		

## Shells with Small Cutouts Testing and Analyses

# **Description of Small Cutouts**

- Small cutouts scaled to represent passenger doors on commercial aircraft fuselage barrel
- Cutouts are 3 in. (axial) x 4.88 in. (circumferential), with 0.50-in. corner radii
- Unreinforced cutouts machined into center of one side of each shell (layup ~ [±45]<sub>2s</sub>)
- 20 back-to-back strain gage pairs around cutout perimeter
- Digital image correlation (DIC) used for full-field visualization



Shell with small cutout installed in test stand

## **Finite Element Models**

- ABAQUS analyses performed
  - Geometrically nonlinear analyses
  - Axial end shortening applied / removed
- S4R shell elements used
  - Acreage elements ~ 0.25 in.-square
  - Each constant thickness and fiber angles
  - Measured IM7/8552 material properties and predicted ply thickness
- Refined FE mesh around small cutouts
  - Parametric mesh refinement studied
  - Refined elements surrounding cutout are
    ~ 0.083 in.-square (1/3 nominal)
  - Refinements have same thickness and layup as parent element



Shell with overlaps laminate thicknesses



# **Finite Element Models (2)**

- ABAQUS analyses performed
  - Geometrically nonlinear analyses
  - Axial end shortening applied / removed
- S4R shell elements used
  - Acreage elements ~ 0.25 in.-square
  - Each constant thickness and fiber angles
  - Measured IM7/8552 material properties and predicted ply thickness
- Refined FE mesh around small cutouts
  - Parametric mesh refinement studied
  - Refined elements surrounding cutout are
    ~ 0.083 in.-square (1/3 nominal)
  - Refinements have same thickness and layup as parent element



Steered ply 3 fiber orientation angles



## Shells with Small Cutouts Shell with Overlaps



+ out / 0 / in -

+ out / 0 / in -

# Shell with Overlaps Test and FEA (Global buckling results shown)





### **Shell with Overlaps Crown & Keel Deflections**

(Prebuckling results shown)





## **Shell with Overlaps Crown Strains**

(Prebuckling results shown)



#### Shell with Overlaps Crown Strains (2) (Postbuckling results shown) **Inner surface** strains not shown 30 Compr. load, 20 klbf **FEA** Analysis Test/DIC 10 **Axial outer** Transv. outer DIC 0 -2 2 -6 -4 0

Strain, millistrain











#### Shell with Overlaps Cutout Top (2) (Postbuckling results shown)



## **Shell with Overlaps Cutout LH Side**

(Prebuckling results shown)









## Shells with Small Cutouts Shell w/o Overlaps



## Shell w/o Overlaps Test and FEA



Shell radial deflections





## **Results Summary**

- Shells with small cutouts evaluated globally and at discrete locations
  - Crown/keel, cutout perimeter
- Loads, deflections and strains measured and computed using nonlinear finite element analyses
- Test-analysis correlation assessed during prebuckling, to global buckling, to stable postbuckling
  - Digital image correlation
  - Displacement transducer
  - Strain gage
- Excellent to very good correlation between FEA and test through buckling, and very good to good correlation at stable postbuckling

## Shells with Large Cutouts Testing and Analyses

# **Description of Large Cutouts**

- Large cutouts scaled to represent cargo doors on commercial aircraft fuselage barrel
- Cutouts dimensions are 8 in. (axial) x 5.25 in. (circumferential), with 0.75-in. corner radii
- Unreinforced cutouts machined into center of one side of each shell (layup ~ [±45]<sub>2s</sub>)
- 24 back-to-back strain gage pairs around cutout perimeter
- 2 displacement transducers centered on vertical edges



Detail of shell with large cutout



# Shell with Overlaps Test and FEA









# Shell w/o Overlaps Test and FEA





+ out / 0 / in -

+ out / 0 / in -



## **Concluding Remarks**

- Structural performance of tow-steered composite shells with cutouts assessed using tests and analyses
- Shells tested in end compression through global buckling, into stable postbuckling, and elastic unloading
- Cutouts cause small reductions in axial stiffness (10%) and global buckling load (15%) vs. shells w/o cutouts
- Geometrically nonlinear finite element analyses performed for detailed comparisons with test results
- Detailed comparisons of local deflections and strains performed from prebuckling to stable postbuckling
- Planning for corresponding detailed comparisons of local behavior for shells with large cutouts

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> Thank you! Questions?

## References

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## **Baseline Shell Structural Performance**

#### (1) Test Results

Prebuckling axial stiffness, klb/in. 1st global buckling load, klbs Postbuckling load, klbs	<b>Shell with Overlaps</b> 531.2 38.8 17.3	<b>Shell w/o Overlaps</b> 328.7 17.2 12.6
(2) I	inear FEA	
Prebuckling axial stiffness, klb/in. 1st global buckling load, klbs	Shell with Overlaps 503.2 37.3	<b>Shell w/o Overlaps</b> 306.7 15.6
(3) Tes	t / Linear FEA	
Prebuckling axial stiffness 1st global buckling load	Shell with Overlaps 1.06 1.04	<b>Shell w/o Overlaps</b> 1.07 1.10

Avg. shell radius, ply thickness and adj.  $E_1$  used. No geometric imperfections.



## Shells with Cutouts Linear FEA

#### (1) Baseline Shells

	Shell with Overlaps	Shell w/o Overlaps
Prebuckling axial stiffness, klb/in.	503.2	306.7
1st global buckling load, klbs	37.3	15.6

### (2) Shells with Small Cutouts

	Shell with Overlaps	Shell w/o Overlaps
Prebuckling axial stiffness, klb/in.	495.2	298.8
1st local buckling load, klbs	19.3	10.4
1st global buckling load, klbs	36.6	15.2

#### (3) Shells with Large Cutouts

	Shell with Overlaps	Shell w/o Overlaps
Prebuckling axial stiffness, klb/in.	491.0	294.7
1st local buckling load, klbs	14.4	8.0
1st global buckling load, klbs	36.3	14.9

Avg. shell radius, ply thickness and adj.  $E_1$  used. No geometric imperfections.



# **Normalized Shells with Cutouts Linear FEA**

#### Performance metric: Cutout / Baseline

#### (1) Baseline Shells

	Shell with Overlaps	Shell w/o Overlaps
Prebuckling axial stiffness, klb/in.	503.2	306.7
1st global buckling load, klbs	37.3	15.6

### (2) Shells with Small Cutouts

	Shell with Overlaps	Shell w/o Overlaps
Prebuckling axial stiffness	0.98	0.97
1st local buckling load*	0.52	0.67
1st global buckling load	0.98	0.97

#### (3) Shells with Large Cutouts

	Shell with Overlaps	Shell w/o Overlaps
Prebuckling axial stiffness	0.98	0.96
1st local buckling load*	0.39	0.51
1st global buckling load	0.97	0.96

Avg. shell radius, ply thickness and adj.  $E_1$  used. No geometric imperfections. \* Divided by baseline 1st global buckling load



## Shells with Cutouts Test Results

#### (1) Baseline Shells

	Shell with Overlaps	Shell w/o Overlaps
Prebuckling axial stiffness, klb/in.	531.2	328.7
1st global buckling load, klbs	38.8	17.2
Postbuckling load, klbs	17.3	12.6

### (2) Shells with Small Cutouts

	Shell with Overlaps	Shell w/o Overlaps
Prebuckling axial stiffness, klb/in.	497.1	299.5
1st local buckling load, klbs	19.9	10.5
1st global buckling load, klbs	31.8	15.5
Postbuckling load, klbs	20.2	10.5

#### (3) Shells with Large Cutouts

	Shell with Overlaps	Shell w/o Overlaps
Prebuckling axial stiffness, klb/in.	488.6	295.6
1st local buckling load, klbs	12.5	7.6
1st global buckling load, klbs	33.0	14.6
Postbuckling load, klbs	20.4	11.5



# **Normalized Shells with Cutouts Test Results**

#### Performance metric: Cutout / Baseline

#### (1) Baseline Shells

	Shell with Overlaps	Shell w/o Overlaps
Prebuckling axial stiffness, klb/in.	531.2	328.7
1st global buckling load, klbs	38.8	17.2
Postbuckling load, klbs	17.3	12.6

#### (2) Shells with Small Cutouts

	Shell with Overlaps	Shell w/o Overlaps
Prebuckling axial stiffness	0.94	0.91
1st global buckling load	0.82	0.90
Postbuckling load	1.17	0.83

#### (3) Shells with Large Cutouts

	Shell with Overlaps	Shell w/o Overlaps
Prebuckling axial stiffness	0.92	0.90
1st global buckling load	0.85	0.85
Postbuckling load	1.18	0.91



# Normalized Test and Linear FEA Results

#### Performance metric: **Test / Linear FEA**

### (1) Baseline Shells

	Shell with Overlaps	Shell w/o Overlaps
Prebuckling axial stiffness, klb/in.	1.06	1.07
1st global buckling load, klbs	1.04	1.10

### (2) Shells with Small Cutouts

	Shell with Overlaps	Shell w/o Overlaps
Prebuckling axial stiffness	1.00	1.00
1st local buckling load	1.03	1.01
1st global buckling load	0.87	1.02

#### (3) Shells with Large Cutouts

	Shell with Overlaps	Shell w/o Overlaps
Prebuckling axial stiffness	1.00	1.00
1st local buckling load	0.87	0.95
1st global buckling load	0.91	0.98