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Defining the Operational Envelope for Air Flows in the miniature Arc-jet Research Chamber (mARC II)

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Motivation





Supporting the development and testing of low-maturity level technologies

NASA Ames has a strong legacy in ground-testing:

- Operates six high-power arc-heaters (10–60 MW) to deliver highenthalpy flows (convective and/or radiative) for extended periods of time and for various gas mixtures
- Relied upon for <u>every</u> NASA mission with entry phase

Demand for a smaller-scale facility to support the rapid, low-cost development of low-<u>maturity technologies</u> prior to them being implemented/tested in the larger facilities:

- Diagnostics (e.g., non-intrusive flow measurements)
- Instrumentation (e.g., intrusive flow measurements)
- Materials (e.g., screening of novel TPS)

ADEPT arc-jet test – 60 MW IHF Credit: NASA/TSF











mARC II's previous vacuum system was unable to N/S/ maintain p_{e}/p_{∞} >1 under test conditions

100 A, 0.25 g/s



- <u>Underexpanded flow</u> (p_e/p_{∞} >1) is desirable to provide large region of constant flow conditions.
- To reliably sustain p_e/p_{∞} >1, the <u>need for a new vacuum system was identified</u>.



<u>High test box pressures</u> after arc-on were the root cause of overexpanded flow $(A/A* \simeq 11.3)$.







mARC II



miniature Arc-jet Research Chamber (second-generation)



Facility — arc heater & test box





<u>Components</u>:

- 1. MAX 200 torch body
- 2. Cathode
- 3. Constrictor disks (x2)
- 4. Anode
- 5. Nozzle
- 6. Test box
- 7. Diffuser (*new*)
- 8. Heat exchanger (*new*)

+ sweep arm for intrusive flow characterization







Facility — flow characterization

Non-intrusive flow diagnostics:

- High-speed camera with notch filter
- Optical emission spectroscopy (OES)

- in progress -

Intrusive flow diagnostics:

- Sweep arm with trident holder
- Water-cooled Gardon gauge (3/16" hemispherical) used for stagnation point heat flux measurements

1 sec dwell ~0.7 m/s sweep speed











Facility — upgraded vacuum system



- direct drive rotary vane pump (both water-cooled).
- Water-cooled diffuser and a heat exchanger were designed in-house and installed.



<u>Refurbished mechanical booster pump</u> was procured and coupled to the existing two-stage





Arc-jet performance

Flow enthalpy equations & numerical modeling







N/S/Flow enthalpy — enthalpy by energy balance (EB^2)

power lost in the arc heater cooling circuit ΔP_{COOL} :

$$\bar{h}_{\text{EB}2} = \frac{P_{\text{arc}} - \Delta P_{\text{COOI}}}{\dot{m}} = \frac{(IV) - (\dot{m}_{\text{W}}c)}{\dot{m}}$$

Commonly used in the <u>high-power arc-jet facilities</u> at NASA Ames.

A bulk enthalpy estimate can be calculated via an <u>energy balance</u> from the arc power P_{arc} and the

$(p_W \Delta T_W)$

where

- I: arc current
- arc voltage V:
- air mass flow rate *m*:
- cooling water flow rate \dot{m}_{W} :
- temperature difference between water $\Delta T_{\rm W}$: supply and return lines
- isobaric specific heat capacity (water) $c_{p,W}$:









N/S/ Flow enthalpy — sonic flow method & correlations

Other formulations are available:

$$\bar{h}^{W} = \left(\frac{123}{\sigma}\right)^{2.52}$$

$$\bar{h}^{S} =$$
Winovich (1964) Shep

The correlation constants are known to be <u>facility dependent</u>.



$$\bar{h}^{\mathsf{T}} = \left(\frac{155.8}{\sigma}\right)^2$$

oard et al. (1993)

Thompson, Prabhu, et al. (2011)

where the sonic flow parameter
$$\sigma = \frac{\dot{m}}{A_t p_{0,COI}}$$
 [kg/m²at







Numerical modeling — laminar Navier–Stokes

<u>Laminar simulations</u> were performed using the Data Parallel Line Relaxation (DPLR) code, v4.04.



boundary conditions.

• Inlet enthalpy \overline{h} was calculated using the Winovich equation (\overline{h}^{W}) .



Experimental measurements for air flow rate \dot{m} , arc column pressure $p_{\rm CO|}$, and test box pressure p_{∞} were combined with chemical equilibrium calculations (CEA) to set the inlet/outlet



Test matrix

Run	Measured arc current I _{arc} [A]	Measured flow rate, ṁ [g/s]
1	42	0.23
2	96	0.23
3	142	0.23
4	188	0.23
5	40	0.23
6	183	0.23
7	42	0.14



<u>Seven runs</u> were completed over the course of a two-day <u>Integrated Systems Testing</u> campaign.

The Gardon gauge was 70 mm from the nozzle exit plane for all conditions.





Results & Discussion







Performance of the new vacuum system



- After arc-on, test box reached steady-state in <10 seconds where p_{α}/p_{∞} >1.
- Average test box pressure during tests: $\sim 17\pm0.2$ Pa (0.13 ±0.002 torr).





2X improvement in pump-down time to base pressure, 13X improvement in base pressure.







Performance of the new vacuum system Change in nozzle flow structure

For the same set conditions: 100 A, 0.25 g/s

Previous vacuum system





New vacuum system







Bulk flow enthalpy EB^2 measurements — $\overline{\dot{m}} = 0.23$ g/s



- Despite low ΔT_W values, <u>SNR(ΔT_W </u>) is high: 29 to 85.

N/S/

We fit linear polynomials through the arc power P_{arc} and cooling power ΔP_{cool} data points.

• A relationship between I_{arc} and h_{FB2} is proposed for $\overline{\dot{m}} = 0.23$ g/s: when a specific enthalpy is desired at this flow rate, the required arc current setting can be simply back-calculated.





Bulk flow enthalpy Comparison with previous mARC datasets



- Using the Winovich equation, we report arc-jet efficiencies of 14 < η < 32%.
- Combination of low current and high flow rate leads to highest η .
- Detailed comparison will be possible after full characterization of the facility is completed.





EB² method: three of the four EB² enthalpy values are lower than those calculated via Winovich.







Stagnation point heat flux 3/16" hemispherical Gardon gauge (water-cooled)





96 A, 0.23 g/s 142 A, 0.23 g/s 41 A, 0.14 g/s 40 A, 0.23 g/s 183 A, 0.23 g/s



on gauge - 68 mm -	•	Upgraded vacuum system has enabed mARC II to generate the designder $\frac{1}{2}$ underexpanded flow (p _e /p _∞ >1).
	•	This has led to a ~ <u>4X reduction</u> in heat [.] for the same set test conditions.
70 mm 		We report the lowest stagnation point h fluxes measured in mARC II to date: – minimum of 26 W/cm ² – maximum of 81 W/cm ²
200		Can deliver the low heating ra

- needed for testing of low-maturity tech (e.g., reusable TPS materials).
- kW/cm² heat fluxes are anticipated for \uparrow currents, \uparrow flow rates, and \checkmark distance from nozzle.











Numerical results Laminar axisymmetric model (42 A, 0.23 g/s)



*Franquet et al., 'Free underexpanded jets in a quiescent medium: A review' (2015).





Radial enthalpy gradient is steep near the walls owing to heat conduction (T_W = 350 K). No visible Mach disk upstream of Gardon gauge for conditions tested in present work*.



Numerical results Stagnation point heat flux — comparing to experimental data









Concluding Remarks & Future Work





Concluding Remarks

- 1. The vacuum system upgrade had a significant impact on mARC II performance:
- 13X improvement in base pressure pre-test & reliable underexpanded flow (p_e/p_{∞} >1) during test.
- Reduced heat fluxes by $\sim 4X$ for same set test conditions.
- Lowest heat fluxes measured in mARC II to date $(26-81 \text{ W/cm}^2)$.
- First demonstration that mARC II can enable research needing low heat rate testing.



Integrated Systems Testing campaign was completed after the mARC II facility was upgraded.



Concluding Remarks

- 2. Bulk enthalpy via EB² method was reported for the first time for mARC II:
- Initial data suggests EB² generally estimates lower enthalpy than sonic flow (e.g., Winovich).
- Simple equation is proposed for interpolating EB² enthalpy based on arc current ($\dot{m} = 0.23$ g/s).
- 3. Laminar numerical simulations were undertaken:
- Good agreement with experimental stagnation point heat flux at low arc currents (3–15%, 42 A). Discrepancy increases with arc power (49%, 188 A).



Integrated Systems Testing campaign was completed after the mARC II facility was upgraded.











Future Work

- 1. Full characterization of mARC II's operational envelope of using air flows will be completed:
 - Stagnation heat flux, radial heat flux profile, and stagnation pressure.
 - EB^2 enthalpy to develop a correlation for the mARC II facility.
- 2. Additional efforts will be undertaken to understand the disparity between experimental and numerical heat flux results at higher set arc currents.
- 3. Optical diagnostics have been implemented to characterize species and shock structures in the flow — further refinements and expansions planned.







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Auxiliary Slides





Bulk flow enthalpy EB^2 measurements — $\Delta \overline{T}_W$ time series signals





- EB² method is driven by the <u>cooling water</u> \bullet <u>flow rate</u> and the <u>temperature change</u> ΔT_W .
- Since water flow rate was measured to be \bullet constant ($\overline{\dot{m}}_W$ = 2.79±0.05 kg/s), we plot the cooling water temperature change with time.
- Signals have been shifted in time so that arc-on times align.
- Despite low ΔT_W values, <u>SNR(ΔT_W) is high</u>: 29 to 85.









Stagnation point heat flux Signal-to-noise in Gardon gauge — 40 A, $\overline{\dot{m}} = 0.23$ g/s









- We sampled at $f_s = 80$ Hz, 500 Hz, and 8000 Hz, depending on the run.
- Four major peaks in the FFT: 300 Hz, 600 Hz, 800 Hz, and 1600 Hz (600 Hz) and 1600 Hz are likely 2nd harmonics).
- Low-pass Butterworth filter used to filter higher frequency noise (AC signals not of interest; rise time of raw signal was matched).
- Filtering does not reduce σ or SNR of signal when $f_s = 80$ Hz, but does at $f_s = 500 \text{ Hz and } f_s = 8000 \text{ Hz}.$
- In future experiments, we will use the highest sampling rate (8000 Hz is currently the maximum) along with the low-pass filter to maximize the SNR of the heat flux measurements.













Test matrix and outputs

	Arc column						Operational envelope (two constrictor disks)								DPLR CFD		
Run	$\dot{m}_{\rm set}$	I _{set}	Arc-on	m	$\bar{I}_{\rm arc}$	\bar{P}_{arc}	$ar{p}_{ m col}$	\bar{p}_{∞}	x _i	$ar{q}$	$\text{SNR}(\bar{\dot{q}})$	$f_s(\dot{q})$	\bar{h}^{W}	$ar{h}_{\mathrm{EB}^2}$	η	$ar{q}$	$ \delta \bar{\dot{q}} $
	[g s ⁻¹]	[A]	[-]	[g s ⁻¹]	[A]	[kW]	[kPa]	[Pa]	[mm]	$[W \text{ cm}^{-2}]$	[—]	[Hz]	[MJ	kg ⁻¹]	[%]	$[W \text{ cm}^{-2}]$	[%]
1	0.25	40	\checkmark	0.23	42	_	15	16	70	53	6.6	80	5.4	_	_	55	3%
2	0.25	100	\checkmark	0.23	96	8.2	18	17	70	68	5.2	80	8.4	9.1	24	85	25%
3	0.25	150	\checkmark	0.23	142	11	20	18	70	73	5.3	80	10.3	_	21	104	43%
4	0.25	200	\checkmark	0.23	188	_	21	18	70	78	3.4	80	11.6	_	_	116	49%
5	0.25	40	\checkmark	0.23	40	4.6	16	17	70	53	38	8000	6.3	5.5	32	_	_
6	0.25	200	\checkmark	0.23	183	15	22	19	70	81	19	500	13.6	10	22	_	_
7	0.15	40	\checkmark	0.14	42	4.1	8.3	13	70	26	6.8	500	4.3	2.7	14	22	15%



