

1 **Production and Characterization of Additively Manufactured Radiator Panels**
2 **with Integral Branching Heat Pipes for High-Temperature Heat Rejection**

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11 **ABSTRACT**

12 Emerging concepts for fission surface power and nuclear electric propulsion necessitate
13 lightweight, mechanically robust, and thermally efficient heat rejection radiators. State-of-the-art
14 intermediate-temperature (~400 K) composite radiator assemblies have been developed based on
15 titanium-water heat pipes bonded to metal, graphite, and carbon-fiber-based panels. NASA has
16 identified a need for new radiator concepts that can operate at even higher temperatures (500 – 600
17 K), minimize thermal resistances and thermal stress failures at bond interfaces, and approach areal
18 densities of 2 – 3 kg m⁻².

19 To meet these needs, our team is developing additively manufactured (AM) radiator panels
20 with integral branching wicking heat pipe networks. Water is selected as the working fluid for this
21 temperature range. Based on simulations and thermal vacuum experiments, these branching
22 embedded heat pipe networks can efficiently distribute heat over panels for finned surface
23 efficiencies of $\eta_f > 70\%$ at $T_H = 500$ K input heat.

24 This paper first presents laser powder-bed fusion AM strategies to produce embedded porous
25 structures for wicking heat pipes in Inconel 718 and titanium alloys (commercially pure and Ti-
26 6Al-4V alloys). Post-build chemical and thermal treatments are described that yield hydrophilic
27 wicking surfaces for operation with water. Transient rate-of-rise experiments with water and
28 acetone are reported that yield estimates for AM wick porosity (ϵ), permeability (K), and effective
29 pore radius (r_{pore}). Based on the wick characterization results, small prototype radiator panels (75
30 × 125 mm) with integrated heat pipe networks were manufactured. Heat rejection performance
31 data are presented from cold thermal vacuum testing, with heat input temperatures up to ~510 K.
32 Future efforts will focus on improving heat pipe performance, optimizing radiator mass, and
33 evaluating larger panels to assess scalability.
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38 **NOMENCLATURE, ACRONYMS, ABBREVIATIONS**

39	A_c (m ²)	Cross-sectional area of the heat pipe
40	g (m s ⁻²)	Earth's gravitational acceleration (9.81 m s ⁻²)
41	h (m)	Water uptake height
42	K (m ²)	Permeability
43	P_{cap} (Pa)	Capillary pressure drop in the wick
44	P_{liq} (Pa)	Liquid pressure drop in the wick
45	P_{net} (Pa)	Net pressure drop in the heat pipe
46	P_{vap} (Pa)	Vapor pressure drop in the heat pipe
47	Q_{rad} (W)	Radiative heat rejected
48	Q_{heater} (W)	Heater power
49	$Q_{loss,heater}$ (W)	Radiative heat loss from the heater block
50	r_{pore} (m)	Pore radius
51	T_{avg} (K)	Radiator surface average temperature
52	T_H (K)	Hot temperature
53	T_{∞} (K)	Ambient temperature of surroundings
54		
55	<i>Greek Letters:</i>	
56	ϵ	Porosity
57	ϵ_1	Emissivity of RUST OLEUM black paint
58	ϵ_2	Emissivity of aluminum.
59	η_f	Fin Efficiency
60	θ (degrees)	Contact angle
61	μ (kg m ⁻¹ s ⁻¹)	Dynamic viscosity of the fluid at ambient temperature

62	ρ (kg m ⁻³)	Density
63	ρ_S (kg m ⁻²)	Areal Density
64	σ_L (kg s ⁻²)	Liquid surface tension
65	σ (W m ⁻² K ⁻⁴)	Stefan Boltzmann constant (5.67·10 ⁻⁸ W m ⁻² K ⁻⁴)

66

67 1. INTRODUCTION

68 Efficient and mass optimized high-temperature radiative heat rejection solutions are central to
 69 emerging fission surface power (FSP) and nuclear electric propulsion (NEP) systems. Mature
 70 spacecraft radiator technologies can operate with $T_H \sim 400$ K at mass per radiative surface area
 71 values of $\rho_S \sim 12$ kg m⁻² (Mason, 2007), although lower ρ_S concepts have been demonstrated at
 72 prototype scale. NASA has recently proposed technical performance targets for high temperature
 73 radiators of: $T_H = 500 - 600$ K with $\rho_S = 2 - 3$ kg m⁻² (Hyers et al., 2013; NASA TechPort, 2023).

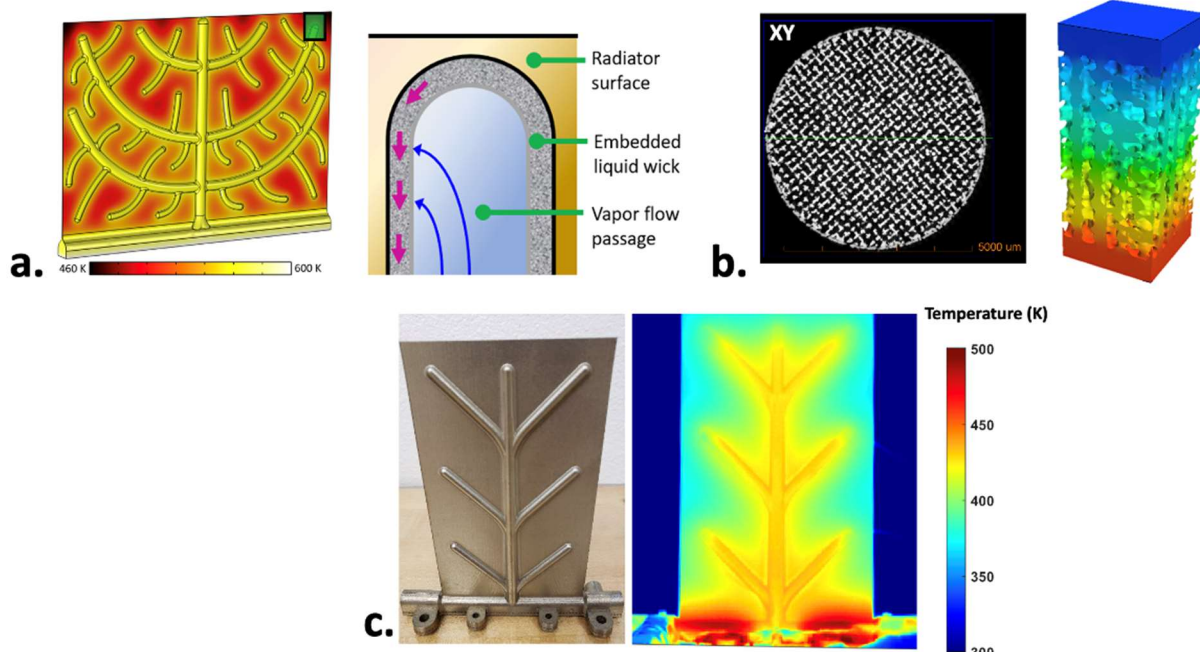
74 A number of research projects have focused on development of mass efficient medium
 75 temperature heat rejection radiators ($T_H \sim 400$ K). A concept for affordable Fission Surface Power
 76 FSP) systems was investigated by NASA and the Department of Energy (DOE) for potential
 77 applications on the Moon and Mars. A candidate heat rejection system consisted of composite
 78 graphite-based radiator panels coupled with titanium-water heat pipes, operating at 400 K and
 79 having an areal density of ~ 3.5 kg m⁻² (Mason et al., 2008). Large panel sections of this design
 80 were validated in thermal vacuum tests. (Tarau et al., 2016) developed and experimentally
 81 characterized a low-cost radiator for FSP applications. Each panel in their design consisted of a
 82 Ti/H₂O thermosyphon adhesively bonded to a graphite fiber-reinforced composite face sheet. The
 83 design, tested at 400 K, showed promising mass efficiency, with an areal density of ~ 1.5 kg m⁻².
 84 However, the polymer adhesive may add significant thermal resistance and limit maximum
 85 temperatures. In a complementary effort targeting $T_H = 400$ K heat rejection, Lee et al., (2020.)
 86 evaluated Ti/H₂O heat pipes with ultrasonically soldered thin aluminum radiative fin panels.
 87 Although that design was successfully able to reject the target heat output of 250 W, it reached a
 88 maximum fin efficiency of 58%. Maximum operating temperatures of such a design could be
 89 limited by the thermal expansion mismatch at the titanium to aluminum bond.

90 Fewer investigations have been reported for higher temperature heat rejection radiators. The
 91 SP-100 Program, a collaboration between NASA, the Department of Energy (DOE), and the
 92 Strategic Defense Initiative Organization (SDIO), began in the 1980s. The goal was to create a
 93 2.5-MWt lithium-cooled, fast-spectrum reactor with highly enriched uranium nitride (UN) fuel
 94 pins to produce a net output of 100 kWe (Sovie, 1987). The heat rejection system concept consisted
 95 of C-C composite panels, and Zirconium alloy heat pipes (Juhasz, 2008). When operating at high
 96 temperature, the radiator faced delamination problems. Similar issues were observed in another
 97 radiator prototype composed of C-C sheets and Ti/H₂O heat pipes (Denham et al., 1994). During
 98 brazing, the composite was reported to fail by delaminating around the braze joint. It could also
 99 break off completely, leaving only a few layers bonded to the titanium.

100 In summary, mass efficient solutions have been demonstrated for medium temperature
101 spacecraft heat rejection radiators. Promising concepts have also been reported for higher
102 temperature operation. However, dissimilar material bonds between heat pipes and radiative panels
103 have generally posed technical challenges. Polymer-based adhesives offer mechanical compliance
104 but have limited working temperature ranges and introduce significant thermal resistance.
105 Specialty brazes can join relevant heat pipe and radiator panel materials, but integrity challenges
106 may occur at high temperatures due to thermal expansion coefficient (CTE) mismatch. Approaches
107 for producing wicking structures in high temperature water heat pipes (*e.g.*, Ti/H₂O for $T_H \gtrsim 500$
108 K) are needed to enable operation against gravity or in micro-gravity environments.

109 To address the needs of FSP and NEP missions, our team is developing fully additively
110 manufactured (AM) radiator panels with integral branched wicking heat pipe networks. Such
111 monolithic heat pipe radiators (HPRs) can be made from high-temperature corrosion-resistant
112 materials, avoiding the mass, thermal resistance, and integrity issues associated with material
113 interfaces and reported in previous designs discussed earlier in this section. Water is selected as
114 the working fluid for operation at $T_H \sim 500$ K. Based on initial simulations and thermal vacuum
115 testing, these embedded branching network heat pipes may efficiently distribute heat over radiator
116 panels, achieving finned surface efficiencies of $\eta_f > 70\%$ at $T_H = 500$ K input heat, and areal
117 density within the desired range of $\rho_S = 2\text{-}3 \text{ kg m}^{-2}$.

118 This paper first presents laser powder-bed fusion AM strategies to produce embedded porous
119 structures for wicking heat pipes (Figure 1a-b) in Inconel 718 and titanium alloys (commercially
120 pure and Ti-6Al-4V alloys). Note that while Inconel 718 may hydrolyze with water at high
121 temperature (Gao et al., 2021), it is used as a test material to develop AM HPRs because of its well
122 documented AM processing characteristics. Post-build chemical and thermal treatments are then
123 described that yield hydrophilic wicking surfaces for operation with water. Transient rate-of-rise
124 experiments with water and acetone are reported that yield estimates for AM wick porosity (ϵ),
125 permeability (K), and effective pore radius (r_{pore}). Prototype small-scale radiator panels
126 (nominally 75×125 mm) were designed with branching heat pipe networks, applying results from
127 these wick characterization experiments (Figure 1c). Heat rejection performance data are presented
128 from cold thermal vacuum testing with heat input temperatures up to $T_H \sim 510$ K.



130

131 **Figure 1:** Scope of reported research progress. **a.** High-level illustration of the heat pipe radiator
 132 design, **b.** Additively manufactured wick development and characterization, **c.** Small-scale radiator
 133 design and manufacturing

134 2. ADDITIVE MANUFACTURING OF POROUS HEAT PIPE WICKS

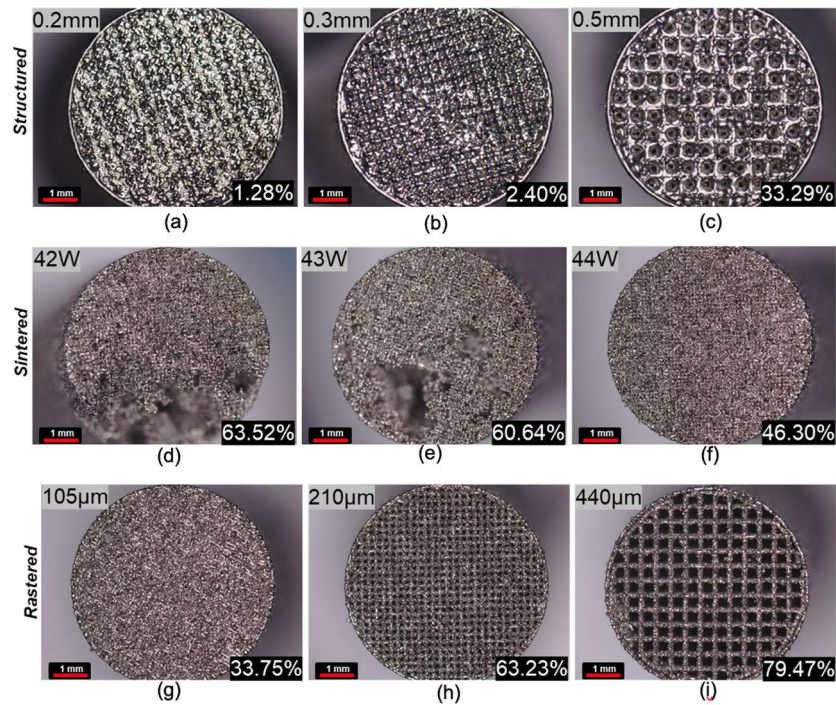
135 2.1 Additive Manufacturing Strategies for Producing Inconel 718 Wicks

136 Three laser powder bed fusion (LPBF) fabrication strategies were explored to produce porous
 137 sections that could serve as wicks for Inconel 718 heat pipes:

- 138 **a. Structured Wicks:** Characterized by deliberate, repeating cell patterns that can potentially
 139 provide precise regulation, structural stability, and advanced features such as size gradients or
 140 local geometry modifications.
- 141 **b. Sintered Wicks:** Created by sintering the metal powder, rather than fully melting it, using low
 142 laser power build parameters. The resulting interconnected porosity between the sintered
 143 particles serves as an area for capillary driven fluid transport.
- 144 **c. Rastered Wicks:** Formed by laser scanning of the powder bed in alternating directions layer-
 145 to-layer to produce interconnected passages. Build parameters have exaggerated hatch spacing
 146 to create a square grid like network of fluid transport channels. The large hatch spacing ensures
 147 open spaces in the lasers raster pattern and prevents melt pool overlap between laser passes.

148 Inconel 718 wick specimens were manufactured as free-standing, vertical cylinders (6 mm
 149 diameter, 40 mm height). Specimens were produced using a GE Additive Concept Laser M2 LPBF
 150 system equipped with a 400 W peak power laser. 120 samples were made using powder feedstock
 151 that had been recycled through sieving 2 – 10 times. Layer height was set at 30 μm, and laser

152 power and scan speed were varied for the different specimens. After fabrication, samples were
153 detached from the build plate, gently brushed, and cleaned with water.



154

155 **Figure 2:** Example images of the different wicks fabricated using the three approaches. Wick
156 porosity is indicated in the lower right corners. In the top left corner of all six figures: For the
157 structured set of samples (a, b, c), the beam thicknesses are indicated. For the sintered set of
158 samples (d, e, f), the different laser powers are mentioned. For the rastered samples (g, h, i), the
159 raster line spacings are specified.

160 2.2 Additive Manufacturing Strategies for Titanium Wicks

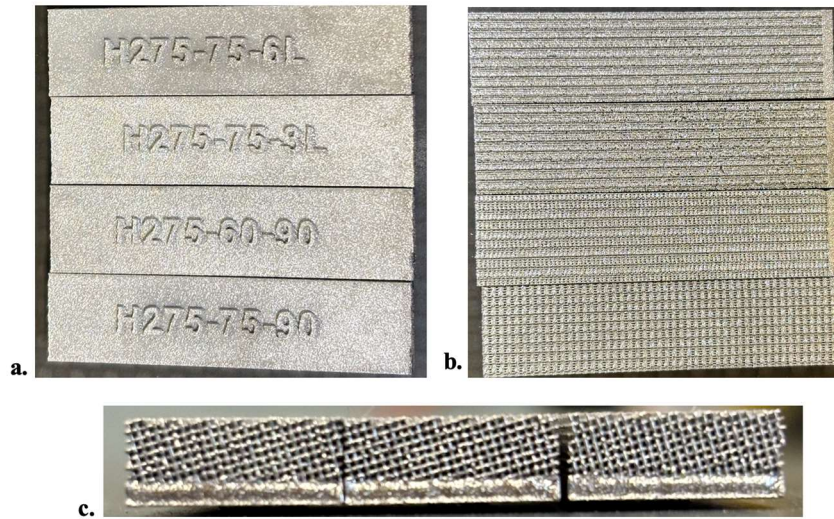
161 Given its chemical compatibility with water (K.-L. Lee et al., 2018), low density
162 ($\sim 4500 \text{ kg m}^{-3}$), and its moderate thermal conductivity ($16 - 20 \text{ W m}^{-1} \text{ K}^{-1}$), commercially pure
163 titanium (CP-Ti) was identified as a candidate material for the AM HPR technology. In preliminary
164 analyses of the Inconel 718 wicks, the rastered approach was found to achieve higher
165 permeabilities, more isotropic flow properties, and fewer defects than the other processing
166 approaches. Therefore, the CP-Ti AM wick specimens were all built with processes based on
167 rastering. To improve permeability, specimens were also produced with raster path rotation after
168 multiple layers rather than every other layer Figure 3. The different raster spacings and patterns,
169 as well as laser power variations and scan speed are recorded in Table 1 Table 1.

170

171 **Table 1:** Printing parameters and raster patterns

Raster Line Spacing (μm)	Rastering Pattern	Laser Power (W)	Laser Scan Speed (mm s^{-1})
275 and 400	Rotating every 1, 3, or 6 layers between X and Y axes	123, 147 & 184	1250

172



173

174 **Figure 3:** a. Labeled wick coupons post-manufacturing. For a hatch spacing of 275 μm , 75% laser
 175 power, and alternating rastering direction every 3 layers between X and Y axes, the coupon label
 176 would be “H275-75-3L”, b. Face view of the wick sticks showing the porous surface,
 177 c. End view of the wick specimens showing the hatch pattern.

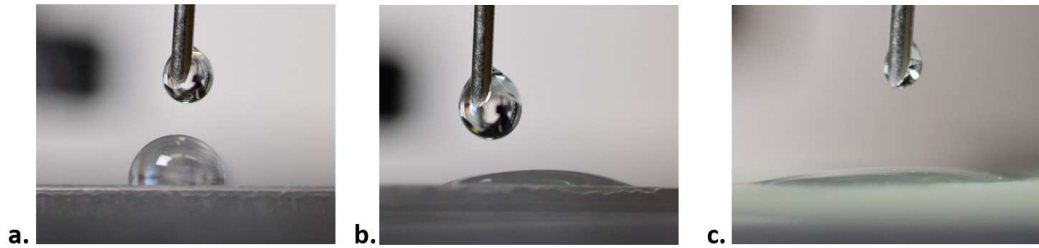
178 As AM CP-Ti builds are not performed routinely, some of the titanium specimens were
 179 produced in more commonly processed titanium alloys (Grades 5 and 23 both Ti-6Al-4V alloys).
 180 Based on guidance from the manufacturing partner, these alloys can be processed with the same
 181 LPBF settings as CP-Ti. As titanium comprises $\sim 90\%$ of these other alloys, it was hypothesized
 182 that they would display similar chemical and wetting properties to CP-Ti.

183 3. WICK TREATMENT TO IMPROVE WETTABILITY

184 3.1 Inconel 718 Oxidation Process to Increase Surface Wettability with Water

185 In the as-built state, Inconel 718 is typically poorly wetting with water, with measured
 186 room-temperature static contact angles of $\sim 70 - 90^\circ$. Many AM specimens did not display
 187 consistent wicking with water in ambient condition tests even though they quickly saturated with
 188 other test fluids (*e.g.*, ethanol and acetone). Based on literature data, Inconel wettability with water
 189 should improve at target working conditions (~ 500 K) (Gojda et al., 2022), but surface
 190 modification to improve wetting could still improve wick performance. The team hypothesized
 191 that intentionally oxidizing Inconel 718 surfaces in heated air environments could evolve
 192 roughened surface features with modified chemistry that could enhance water wettability. Based

193 on literature data for oxidation rates of Inconel 718 (Greene & Finfrock, 2000), a test campaign
194 was conducted in which plain specimens were heated in air at 1000°C for up to 2 hrs. Based on
195 mass changes, the resulting oxide films were believed to be $\leq 1 \mu\text{m}$ thick. Sessile water droplet
196 tests were performed to measure static contact angles of different specimens (Figure 4). It was
197 found that a 40 min thermal oxidation treatment yielded nearly complete wetting, which could
198 translate to high performing AM wicks. Longer heat treatments caused significant surface damage
199 and are not recommended. This oxidation process was applied to all Inconel 718 wick specimens
200 before characterization with water for rate of rise tests (see Section 4).



201
202 **Figure 4:** Representative images from sessile droplet static contact angle tests of water on Inconel
203 718 at room temperature. **a.** Droplet on untreated surface. **b.** Surface oxidized in air at 1000°C for
204 20 minutes. **c.** Surface oxidized in air at 1000°C for 40 minutes.

205 3.2 Titanium Processing to Increase Surface Wettability with Water

206 Titanium has been documented to rapidly passivate in most environments, leading to degraded
207 wettability (Long et al., 2022). Limited research is available on improving hydrophilic properties
208 of titanium specifically for heat pipes. However, titanium medical implants are widely used, and a
209 number of strategies have been reported to increase surface activity for improved biocompatibility
210 and cell adhesion. Some studies explored simple air-atmosphere heat treatment, such as the work
211 of Toffoli *et al.* (2020), in which specimens were heated to 400 – 750°C. Similar to our approach
212 for Inconel 718 specimens, this process was claimed to improve surface wettability by growing a
213 layer of anatase TiO_2 (Kuroda et al., 2023). Other studies evaluated alkaline solution treatments
214 with NaOH and KOH, followed by high temperature heat treatment (Kazek-Kęsik et al., 2017;
215 Kim et al., 2013; Lim et al., 2001). These treatments may generate thin hydrophilic layers
216 composed of sodium or potassium titanates and TiO_2 (Kazek-Kęsik et al., 2017). Previous studies
217 evaluated chemical oxidation with hydrogen peroxide (5 – 25% concentration) followed by heat
218 treatment at 600°C (Khodaei et al., 2020; Khodaei & Hossein Kelishadi, 2018). It should be noted
219 that the studies of Khodaei et al. did not report H_2O_2 concentration changes during treatments. As
220 H_2O_2 can rapidly decompose at elevated temperatures, solution concentrations may have varied
221 during such tests. Many studies performing chemical treatment of titanium surfaces also applied a
222 final heat treatment step. However, the purpose of the heat treatment was not always clearly
223 identified. (Karthega et al., 2010) argued that this thermal treatment primarily modifies the TiO_2
224 layer morphology to improve durability, rather than growing additional oxide.

225 A number of treatments were tested based on this literature to improve and preserve water
226 wettability of the titanium wick specimens (see Table 2). Simple air-atmosphere heat treatment
227 (1 hr at 400°C, 500°C, 550°C, 600°C) resulted in hydrophilic behavior immediately after cooling

228 for all specimens. However, this enhanced wettability diminished over several days in the
 229 laboratory environment. Alkaline solution treatments were applied to CP-Ti specimens at various
 230 temperatures, durations, and temperatures, followed by heat treatment at 600°C for 1 hr. These
 231 treatments resulted in very hydrophilic surfaces, but precipitates were visible in wicks, even after
 232 multiple washing cycles in an ultrasonic cleaning system. These precipitates could block flow and
 233 cause corrosion over time and would be even more difficult to remove from a long branching heat
 234 pipe network.

235 Chemical oxidation treatment was evaluated with 15wt% H₂O₂ solution at 60°C for 4 hours,
 236 followed by thermal treatment in air atmosphere at 600°C for 1 hour. This produced very
 237 hydrophilic CP-Ti wick specimens. The resulting hydrophilic property was found to persist for
 238 multiple months in the laboratory environment. As the residual products of the H₂O₂ reaction with
 239 titanium should only be water, oxygen and hydrogen gases, blockage of wick pores and long-term
 240 corrosion are not anticipated. Based on this performance and these considerations, the H₂O₂
 241 treatment, followed by air-atmosphere heat treatment were adopted for subsequent development
 242 of CP-Ti HPRs.

243 **Table 2:** Evaluated treatments to improve water wettability of additively manufactured titanium
 244 wick specimens and results. Treatment described in last row (bold) was adopted for subsequent
 245 CP-Ti heat pipe radiator experiments.

Treatment	Tested Materials	Results
Heat Treatment in air atmosphere (~1 hr) at: 400, 500, 550, 600, & 750°C	CP-Ti Gr. 5 Ti Gr. 23 Ti	Increased wettability, but only temporary. Hydrophilic properties degraded within days.
NaOH chemical treatment, followed by heat treatment at 600°C	CP-Ti	Increased wettability, but large residual deposits, potentially plugging wick pores
KOH chemical treatment, followed by heat treatment at 600°C	CP-Ti	
H₂O₂ chemical treatment, followed by heat treatment at 600°C	CP-Ti Gr. 5 Ti Gr. 23 Ti	Increased wettability for CP-Ti specimens, with slower degradation in wettability over time in laboratory environment. Gr. 5 and Gr. 23 specimens became more hydrophobic.

246

247 **4. WICK CHARACTERIZATION: POROSITY, PERMEABILITY AND PORE RADIUS**
 248 **ESTIMATION**

249 **4.1 Rate of Rise Experimental Set-Up and Results for AM Inconel 718 Wick Specimens.**

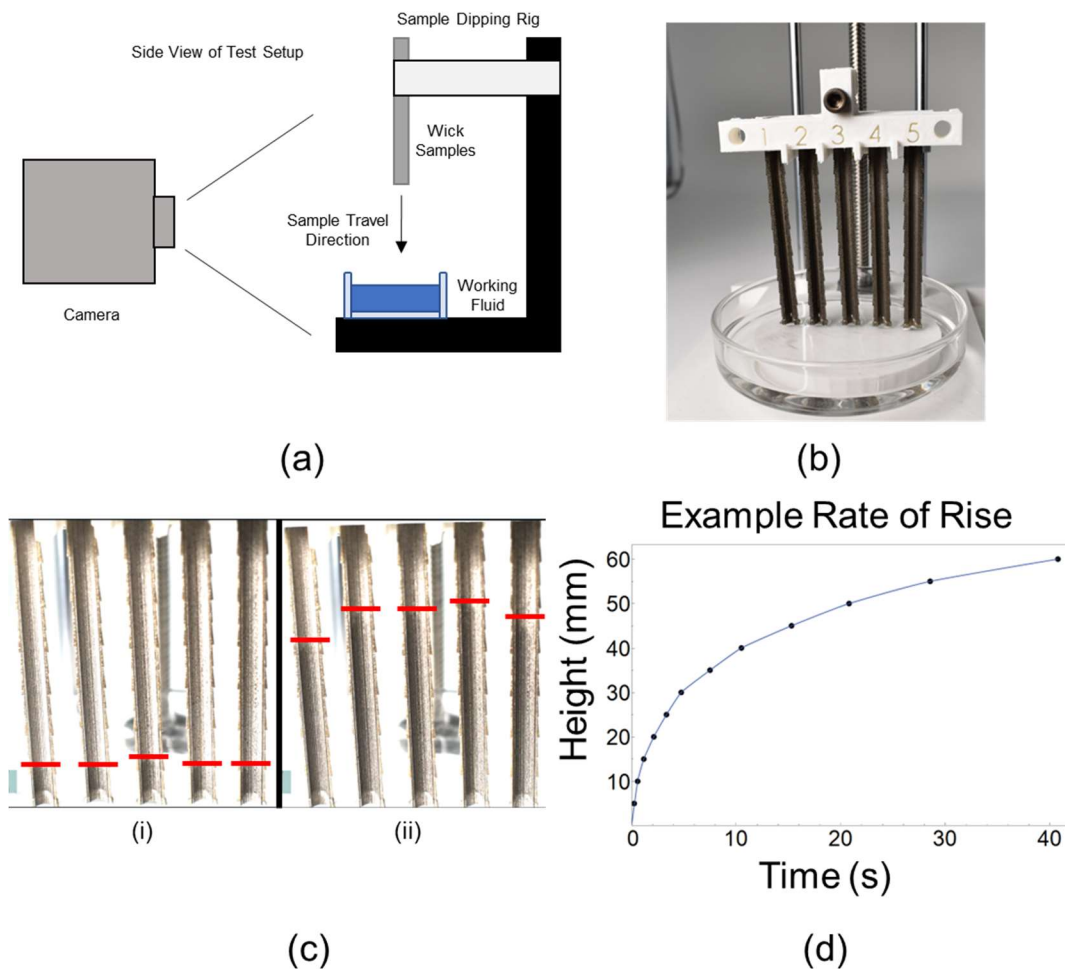
250 To evaluate capillary performance of Inconel 718 wicks, dry specimens were dipped, in groups
 251 of five, into a small pool of working fluid (Deionized (DI) water, and ethanol) using a manually
 252 operated dipping rig (Figure 5a). Fluid uptake was recorded on a Nikon D3500 camera for
 253 approximately 60 seconds. Videos were examined frame by frame to identify timestamps for when

254 the fluid front passed the 5 mm ridges on the sample edges (Figure 5Figure 5b-c). Rate-of-rise
 255 curves were thus collected for Inconel 718 samples in both the as printed condition and after
 256 thermal oxidation. These curves Figure 5(Figure 5d) were then used to fit permeability (K) and
 257 average pore radius values (r_{pore}) with a theoretical model for capillary rise (Equation 1).

258
$$\frac{\rho \epsilon}{2} \frac{d^2 h(t)}{dt^2} = \frac{2 \sigma_L \cos \theta \epsilon}{r_{pore}} - 2 \frac{\mu}{K} h(t) \frac{dh(t)}{dt} - \rho g \epsilon h(t) \quad (1)$$

259 This model balances fluid inertia (left hand side) with capillary pressure (first term on right),
 260 Darcy flow resistance (second term on right), and gravitational head (last term). Contact angles
 261 (θ) are estimated as 10° for ethanol (and later acetone for CP-Ti wicks) and 15° for water for
 262 oxidized wicks.

263

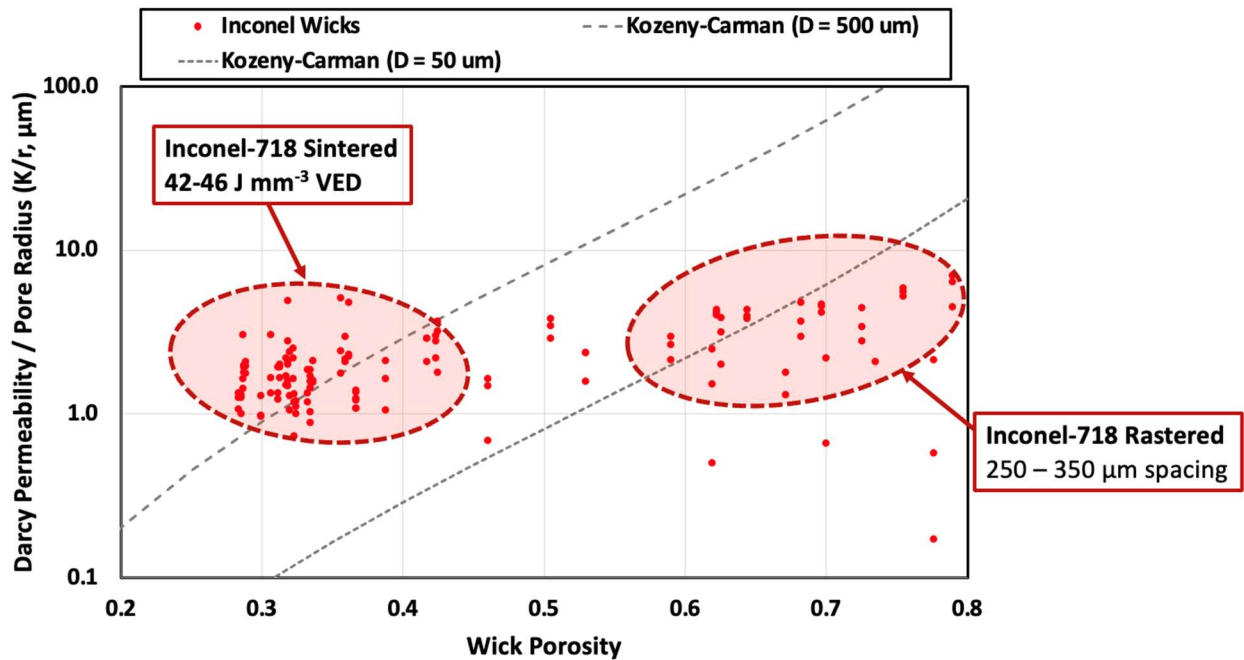


264

265 **Figure 5:** (a) Schematic of dipping rig setup, (b) Image of dipper setup with samples dipped into
 266 working fluid, (c) Images adjusted in software to better visualize the wick fluid front at two
 267 different times. Red lines indicated top of fluid front (i) Time: 3.83 s and (ii) Time: 17.88 s. (d)
 268 Data extracted from the video frames is compiled to create a rate of rise curve for each sample.

269 Example of a rate of rise curve for rastered wick sample: Wick Thickness: 0.4 mm, Print
270 Orientation: Vertical, Working Fluid: Ethanol.

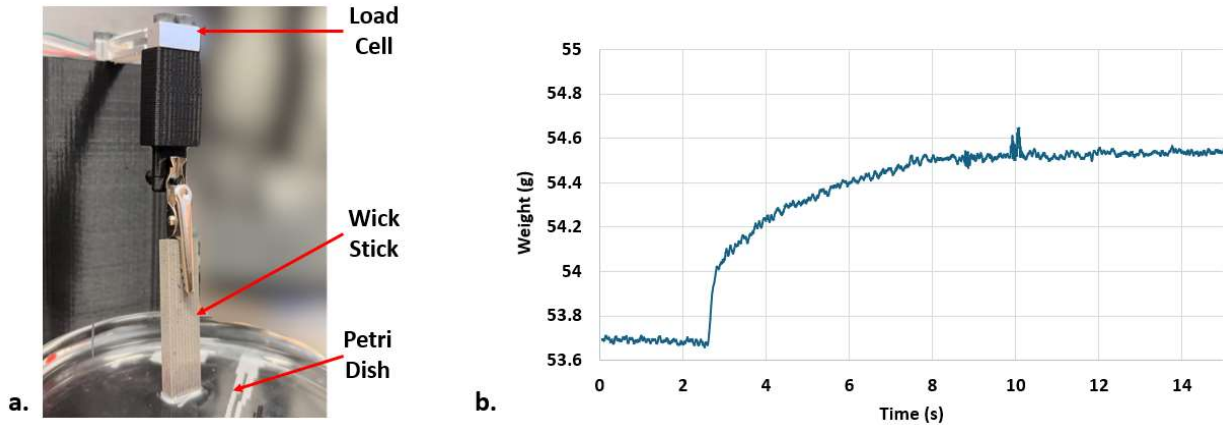
271
272 Rastered wicks with large hatch spacings ($>250 \mu\text{m}$) printed in the vertical orientation
273 produced the greatest capillary performance ratios (K/r_{pore}). Oxidation of the Inconel wicks
274 greatly improved the wettability of the wick surface when using water working fluid. High
275 performing rastered wick specimens had porosity ranges of 58 – 80%, permeabilities of
276 $100 - 440 \mu\text{m}^2$, and average pore radius values of $42 - 168 \mu\text{m}$ (Figure 6). Some sintered wicks
277 yielded good performance, with K/r_{pore} values slightly below the best rastered wick values.
278 However, sintered wicks tended to have more anisotropic permeabilities.
279



280
281 **Figure 6:** Darcy permeability to pore radius ratio (K/r_{pore}) of oxidized Inconel 718 sintered and
282 rastered wick specimens plotted with respect to porosity. Reported values were measured from
283 room temperature rate-of-rise experiments with water.

284
285 **4.2 Rate of Rise Experimental Set-Up and Results for AM Titanium Wick Specimens.**

286 A transient weight-based rate-of-rise setup was prepared to estimate the effective
287 permeabilities (K) and average pore radius values (r_{pore}) of titanium wick specimens. The
288 specimens were rectangular shaped, having a length of 40 mm, a width of 10 mm and a thickness
289 of 3 mm, 2 mm of which are porous. In these tests, each wick specimen was suspended from a
290 high-sensitivity load cell over a small pool of a test liquid (acetone or deionized water, Figure 7a).
291 Liquid was gradually added to the pool until the interface contacted the base of the wick. Transient
292 weight measurements were collected as liquid wetted and climbed the wick specimens (Figure 7b).



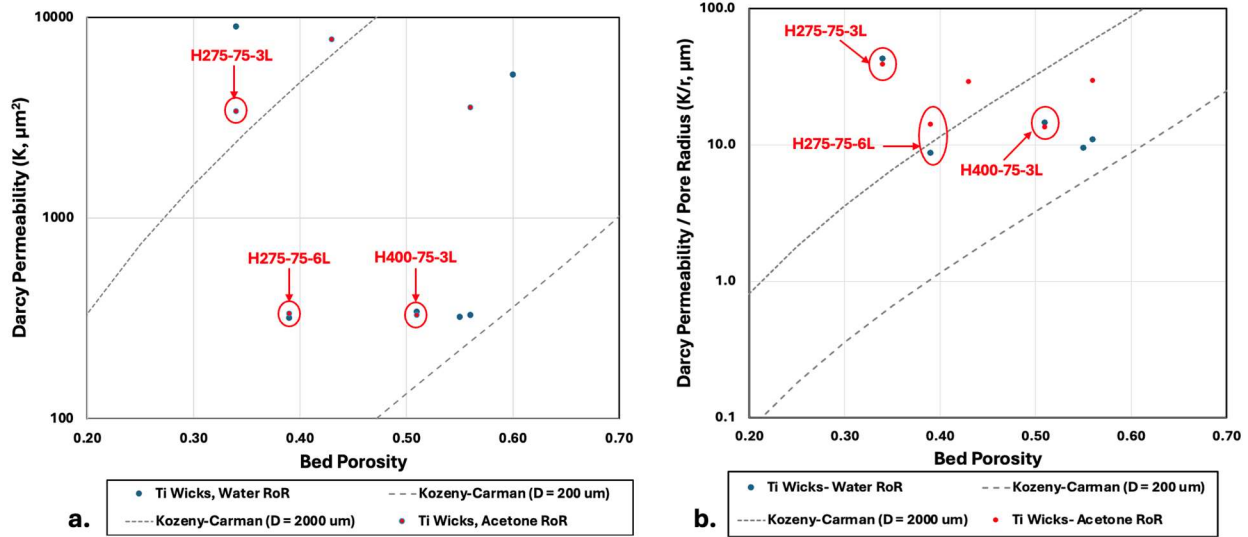
293
 294 **Figure 7:** a. Benchtop rate-of-rise test rig, b. Representative load cell readings during a rate-of-
 295 rise experiment rate for a wick specimen.

296
 297 Transient weight measurements were smoothed to reduce signal noise. Liquid uptake weight
 298 (mg) was converted into liquid height based on separate porosity measurements (mass-based) and
 299 assuming uniform liquid rise height through the wick cross-section. Porosity values for each wick
 300 coupon tested were obtained by first estimating the wick and solid regions volume. Using the dry
 301 mass of the coupon, along with the volumes of the wick and solid regions, the porosity for each
 302 specimen was calculated. The rate-of-rise momentum balance model (Equation 1) was then fit to
 303 measured data by minimizing integral RMS error for effective permeability (K) and average pore
 304 radius (r_{pore}). Permeability and pore radius values are summarized for titanium specimens tested
 305 using acetone and water (Table 3). Note that there is significant variability between the water and
 306 acetone measurements for K and r_{pore} for the H275-75-3L and
 307 H400-60-90 specimens obtained from this combined fitting approach. Overall performance
 308 measures (K/r_{pore}) are less affected. Additional repeatability tests are planned in the future to
 309 improve these estimates.

310 **Table 3:** Representative calculated properties of Ti wick coupons from water and acetone rate-of-
 311 rise test results. Here, contact-angle (θ) is estimated as 15° for water and 10° for acetone.

Coupon	Porosity (ϵ)	Permeability ($K, \mu m^2$)	Pore Radius ($r_{pore}, \mu m$)	K/r_{pore} (μm)
H275-75-3L (Water)	34%	8970	209	43
H275-75-3L (Acetone)		3390	87	39
H275-75-6L (Water)	39%	320	36	9
H275-75-6L (Acetone)		330	24	14
H400-60-90 (Water)	56%	330	30	11
H400-60-90 (Acetone)		3550	120	30
H400-75-3L (Water)	51%	340	23	15
H400-75-3L (Acetone)		330	24	14

313 Darcy permeability (K) and permeability to pore radius ratio (K/r_{pore}) values for the CP-Ti
 314 wicks are summarized in Figure 8. Best performance was obtained with rastered wicks
 315 (275 – 400 μm hatch spacing) produced by rotating the rastering direction (along X- or Y-axis)
 316 every 3 build layers. These settings were adopted for full CP-Ti HPR builds. Correlation curves
 317 for properties of sintered powder wicks of comparable length scales are included for reference
 318 (Costa, 2006).



319 **Figure 8:** Measured AM titanium wick permeability (a) and permeability-to-pore radius (b)
 320 compared with analytic model values for packed beds of spheres of comparable length scales. The
 321 wicks with the most consistent results are highlighted and labeled in red.
 322

323 5. HEAT PIPE RADIATOR DESIGN, PRODUCTION, AND TESTING

324 5.1 Design of Prototype Heat Pipe Radiator Specimens

325 Prototype AM Ti radiator panels with integral branching heat pipes were produced with high
 326 performing internal rastered wick structures described in Sections 2-4. The main panel sections of
 327 the radiators were 75×125 mm with a fin web section thickness of $500 \mu\text{m}$ (Figure 9a-b). Four
 328 mounting tabs on the base are used to attach an external heater unit. Two outer tabs are used to
 329 suspend the HPR and heater assembly on thin threaded rods to minimize conduction losses during
 330 thermal vacuum testing. A compact manifold assembly in the base allows charging of working
 331 fluid, vacuum of non-condensable gases, and sealing during testing.

332 The HPR specimens have a horizontal evaporator section on the base, which connects to a
 333 vertical central trunk that branches into 6 angled sections. A double-pipe geometry is employed in
 334 the horizontal evaporator to increase heat transfer area and reduce the risk of critical heat flux or
 335 boiling-limited dry-out (

336 Figure 9a-b). The overall layout and spacing of branches were selected to achieve radiative fin
 337 surface efficiency of $\eta_f \sim 75\%$ assuming isothermal heat pipes at ~ 500 K. Based on the findings

338 discussed in Section 4, the best working wicking parameters were implemented in the small-scale
339 CP-Ti radiator sheets and summarized in Table 4.

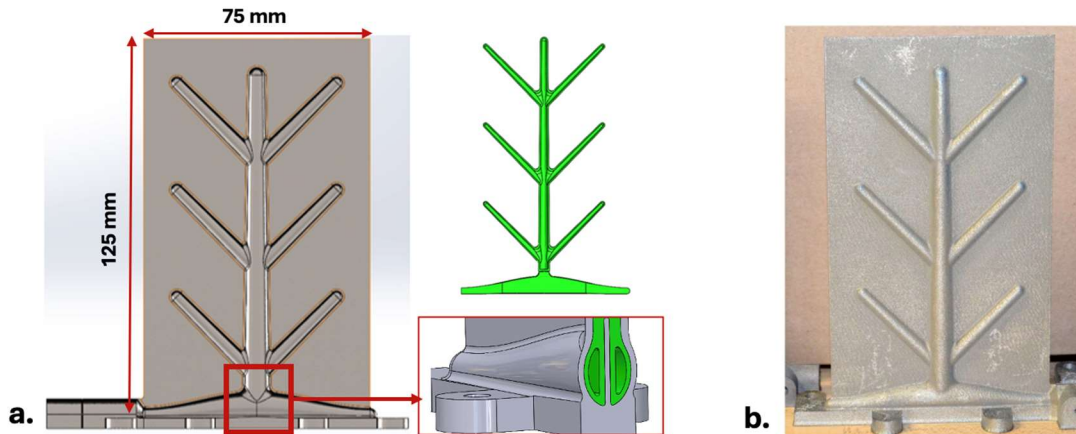
340

341

342 **Table 4:** Summary of prototype small-scale heat-pipe radiator dimensions and wick parameters.

Parameters	Values
Total Height of the design (mm)	131.20
Total Width of the design (mm)	109.53
Height of the radiator fin (mm)	125
Width of the radiator fin (mm)	75
Fin Thickness (μm)	500
Wick Thickness Range (mm)	0.49 (thin branches) – 0.95 (centerline branch)
Vapor Diameter Range (mm)	1.50 (thin branches) – 4.64 (centerline branch)
Theoretical Panel Mass (g)	87.5
Wick Raster Line Spacing (μm)	275 and 400
Raster Strategy	Alternating printing directions between X and Y axes every 3 or 6 layers

343

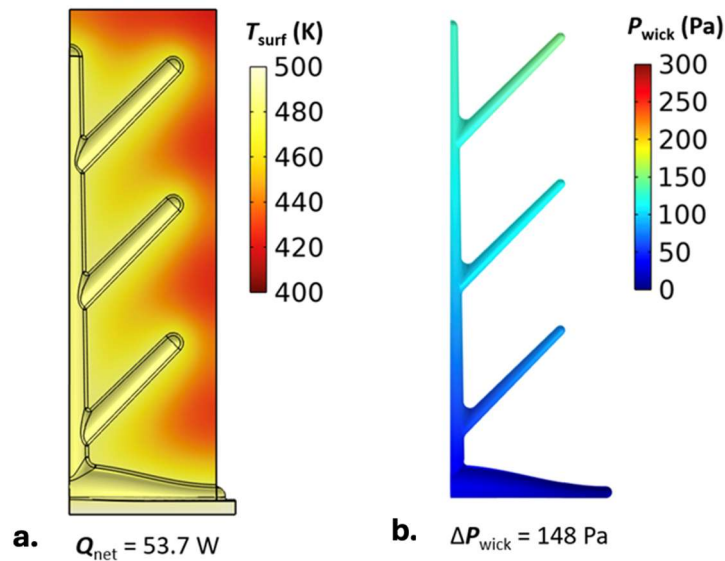


344

345 Figure 9: a. CAD drawing of the radiator panel, the wick in green and a cross-sectional view of
346 the evaporator region, b. Produced radiator panel.

347 Multiphysics simulations were conducted to guide this design accounting for solid domain
348 conduction, vapor flow in the internal passages, porous media transport in the wick (with
349 experimentally measured permeability and pore radius values), liquid-vapor phase change at the
350 wick interface, and external radiation (Figure 10). Vapor flow and wick cross-sections were
351 adjusted based on simulations for this design so that the total predicted pressure range in the heat
352 pipe network: $\Delta P_{net} = \Delta P_{wick} + \Delta P_{vapor} \sim 200$ Pa. This is safely below the expected capillary
353 pumping capacity of $\Delta P_{cap} = 2\sigma/r_{pore} \sim 630$ Pa at temperature and should therefore avoid local
354 wick dry out. The employed permeability ($K = 2000 \mu\text{m}^2$) and a pore radius ($r_{pore} = 100 \mu\text{m}$)
355 values were representative of measurements described in Section 4.2.

356

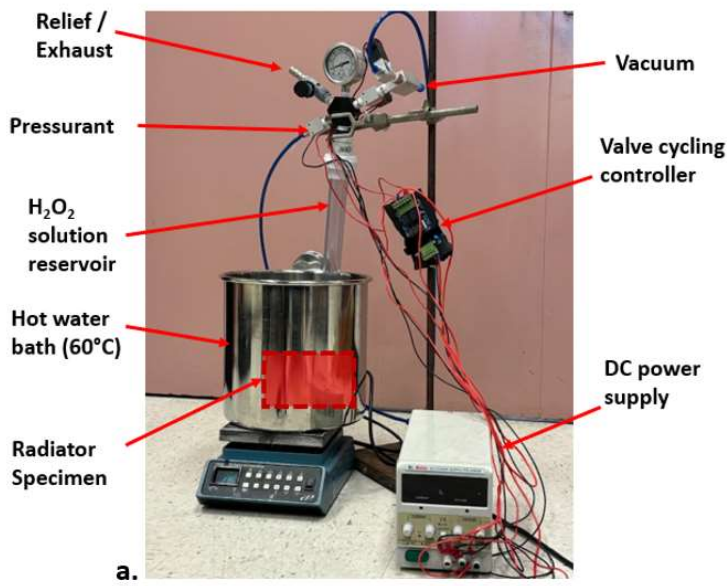


357 **Figure 10:** Representative simulation predictions for: **a.** External surface temperature and
358 **b.** Wick pressure drop for CP-Ti radiator (quarter symmetry section).

359 5.2 Post-Build Processing of Radiator Prototypes

360 After LPBF manufacturing, the HPR specimens were mechanically de-powdered. A cyclic
361 batch chemical treatment system was built to apply the H₂O₂ solution treatment to the interior of
362 the branching heat pipes network. So far, one specimen has been processed, which has a wick
363 raster line spacing of 275 μm with scan direction rotating every 3 layers. In this process, 15% H₂O₂
364 solution was forced into the heat pipes with ~600 kPa absolute air-overpressure for 2-minute
365 intervals. A check valve on the solution reservoir allowed continuous venting of gas to prevent H₂
366 and O₂ buildup from the hydrolysis and H₂O₂ decomposition reactions. Then, vacuum would be
367 applied (to ~10 kPa absolute) for ~10 s to remove evolved gases from the heat pipe network. This
368 approach was developed to ensure that solution reaches the full internal heat pipe network, and
369 that gas build-up does not displace liquid solution, and prevent local oxidation (Figure 11a).
370 During treatment, the HPR specimen (Figure 11b) was submerged in a heated water bath to
371 maintain temperature at ~60°C. After solution treatment, the radiator was flushed with deionized
372 water and heat treated at 600°C in an air atmosphere furnace for one hour.

373 Finish machining operations (porting, lapping heated based) were then performed. The radiator
374 was media blasted to remove scale and painted flat black with a high-emissivity spray paint
375 (RUST-OLEUM specialty high heat; $\epsilon \sim 91-94\%$ (Bellamy et al., 2023)). The paint was thermally
376 cured per the manufacturer specification.



377

378 **Figure 11:** a. Annotated picture of set-up for the hydrogen peroxide oxidation treatment, b. Picture
 379 of an untreated CP-Ti small-scale radiator.

380 5.3 Thermal Characterization Test Facility and Procedure

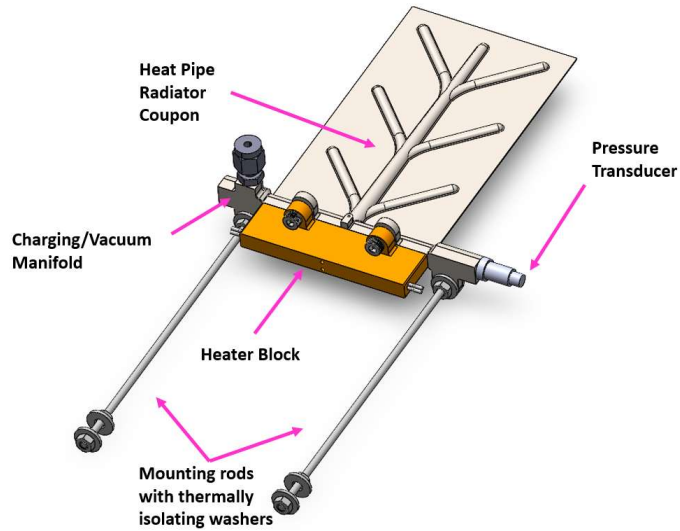
381 A copper heater block was mounted to the radiator base with thermal contact facilitated by a
 382 specialty high-temperature vacuum-rated thermal compound (silver-loaded Aremco Heat-Away
 383 641-EV). The copper heater was loosely wrapped with a couple layers of aluminum foil to limit
 384 radiative heat loss. The HPR and heater assembly were mounted on two long 4 mm diameter
 385 threaded rods to limit conductive heat loss (Figure 12).

386 The HPR was installed horizontally (large faces approximately parallel to ground) in a liquid-
 387 nitrogen (LN2) cooled sub-enclosure inside a vacuum chamber (Figure 13). This approximates
 388 neutral-gravity operation in a cold space environment. The inside of the sub-enclosure enclosure
 389 was coated with Aeroglaze Z306, a high-vacuum-rated paint with a high emissivity of
 390 approximately 0.95 (McCroskey et al., 2000).

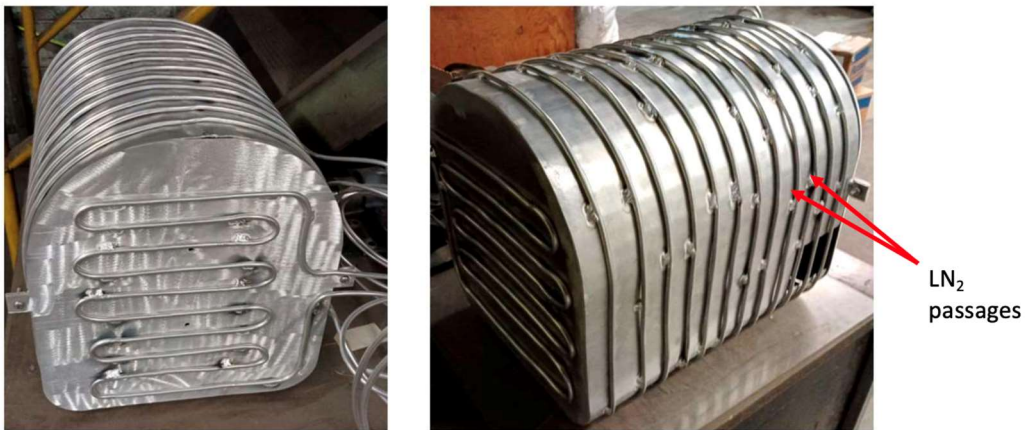
391 The copper heater block was regulated at specified temperatures with a proportional output
 392 PID process controller (Dwyer Love Controller 16B). An AC power transducer (CR Magnetics
 393 CR6210-150-1) was used to measure instantaneous heater power (± 1 W uncertainty). A LabJack
 394 T7 data acquisition unit was employed to collect thermocouple and heater power measurements.

395 Sealed thermocouple feedthroughs were used to measure surface temperatures on 9 points
 396 around the LN2 cooled sub-enclosure, the copper heater block, and two points on the underside of
 397 the HPR specimen. A vacuum-compatible, miniature long-wave infrared thermal imaging camera
 398 (FLIR Boson 320 60Hz) thermal imaging camera was used to measure surface temperature
 399 distributions on the top face of the radiator (Figure 14).

400

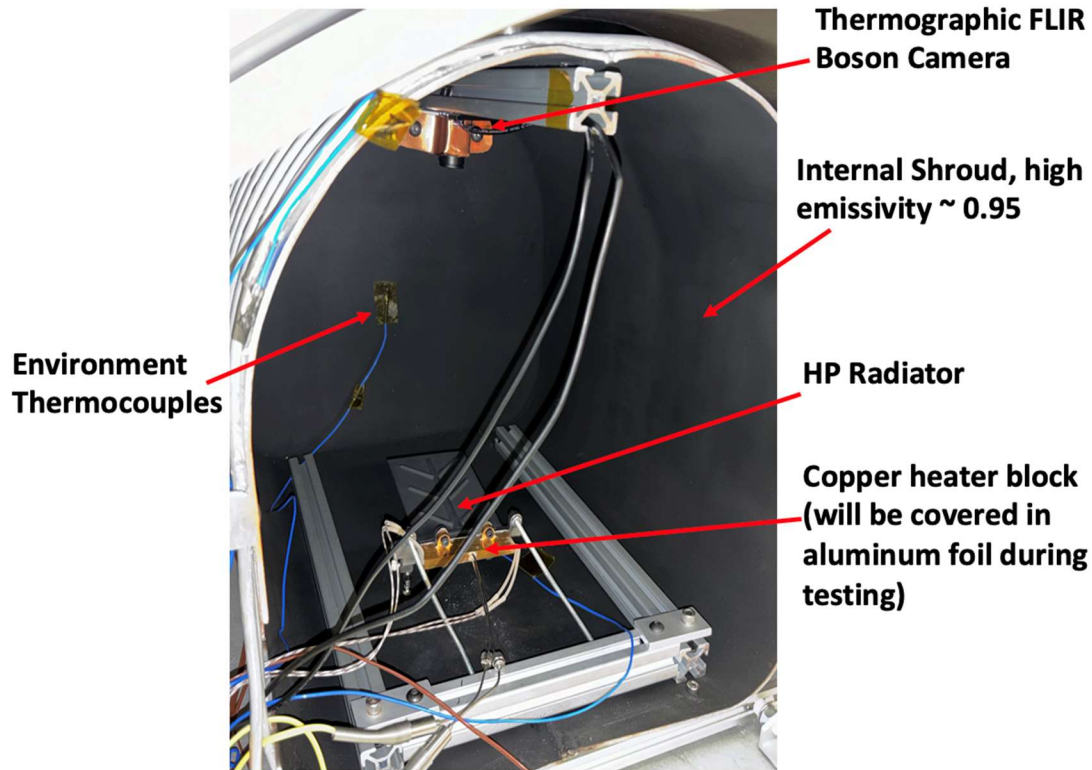


401
 402 **Figure 12:** Rendering of HPR assembly for thermal vacuum testing. Note, the final CP-Ti design
 403 did not include a pressure transducer port.



404
 405 **Figure 13:** Aluminum sub-enclosure featuring the cooling LN₂ tubing. Note that the final installed
 406 unit has nearly continuous solder applied along cooling tube lines.

407 The test platform was installed in the vacuum chamber and verified for proper operation with
 408 cryogenic coolant supply. This facility was successfully employed to perform a set of vacuum
 409 characterization studies for the HPR specimens with heat inputs up to 508 K (235°C). The chamber
 410 reached a vacuum pressure of 10^{-5} torr ($\sim 1.3 \cdot 10^{-3}$ Pa).



411

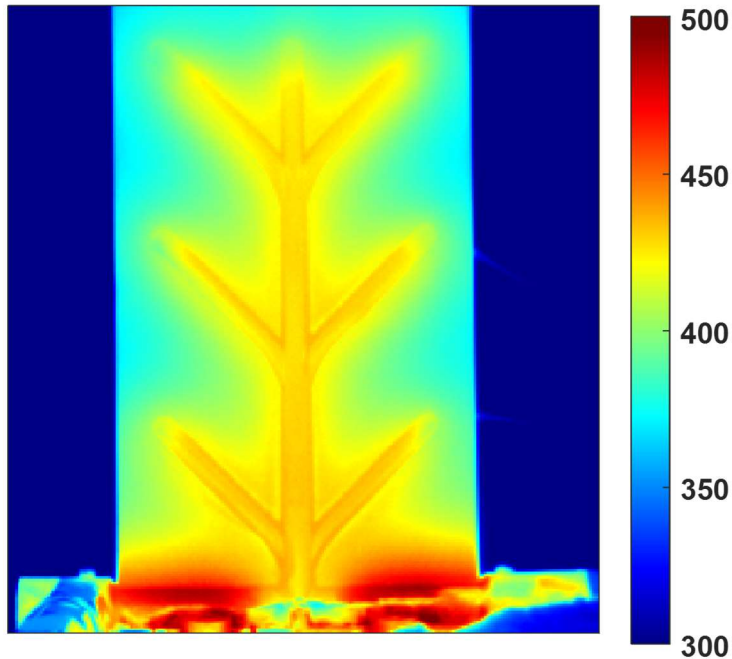
412 **Figure 14:** Labeled diagram of heat pipe radiator test facility, featuring the shroud, heater block
 413 assembly and the thermographic camera installed in the thermal vacuum chamber.

414 To charge the radiator, deionized water was injected into the heat pipe network using a syringe
 415 equipped with a fine-gauge needle. The radiator was heated to a moderate temperature (~50°C).
 416 Vacuum was then cyclically applied to the radiator through a valve to remove non-condensable
 417 gases and reduce the liquid charge to an appropriate level. After each vacuum cycle, the radiator
 418 was inspected with thermal imaging to determine whether non-condensable gases had been
 419 removed (corresponding to large temperature variations in the evaporator) and whether excess
 420 fluid charge was present (corresponding to flooded, non-isothermal heat pipe branch tips). This
 421 sequence was repeated until the entire heat pipe network was fully activated and exhibited an
 422 isothermal condition.

423 5.4 Thermal Characterization Results for Titanium Heat Pipe Radiator

424 Thermal imaging results are presented for the radiator test in Figure 16 for heater block
 425 temperatures of $T_H = 373, 423, 473,$ and 508 K. Overall, the thermal imaging results showed
 426 successful operation of the heat pipe network without evaporator dry out.

427 In earlier stages of the project, the team investigated Inconel 718 radiator panels produced with
 428 different printing strategies. Thermal vacuum testing of those coupons showed significant dry-out
 429 of the evaporator at working temperature. This can be a result of low permeability or defects in the
 430 wick structure. It may also be that water charge was not optimal during the tests (Figure 15).



431

432 **Figure 15:** Vacuum testing thermal imaging results for previously investigated Inconel 718
 433 radiator panel. Condenser portion of the heat pipe network is relatively isothermal, but there is
 434 significant dryout in the lower evaporator.

435

436 In thermal images featured in Figure 16b-d, the cryogenically cooled background temperature
 437 is reported as uniform at 119.9 K. This is slightly below the average surface thermocouple
 438 temperature measurements of 139 – 145 K. It is suspected that the thermal imaging camera
 439 firmware applies thresholding to such cryogenic temperature readings, but the manufacturer has
 440 not responded to requests for documentation.

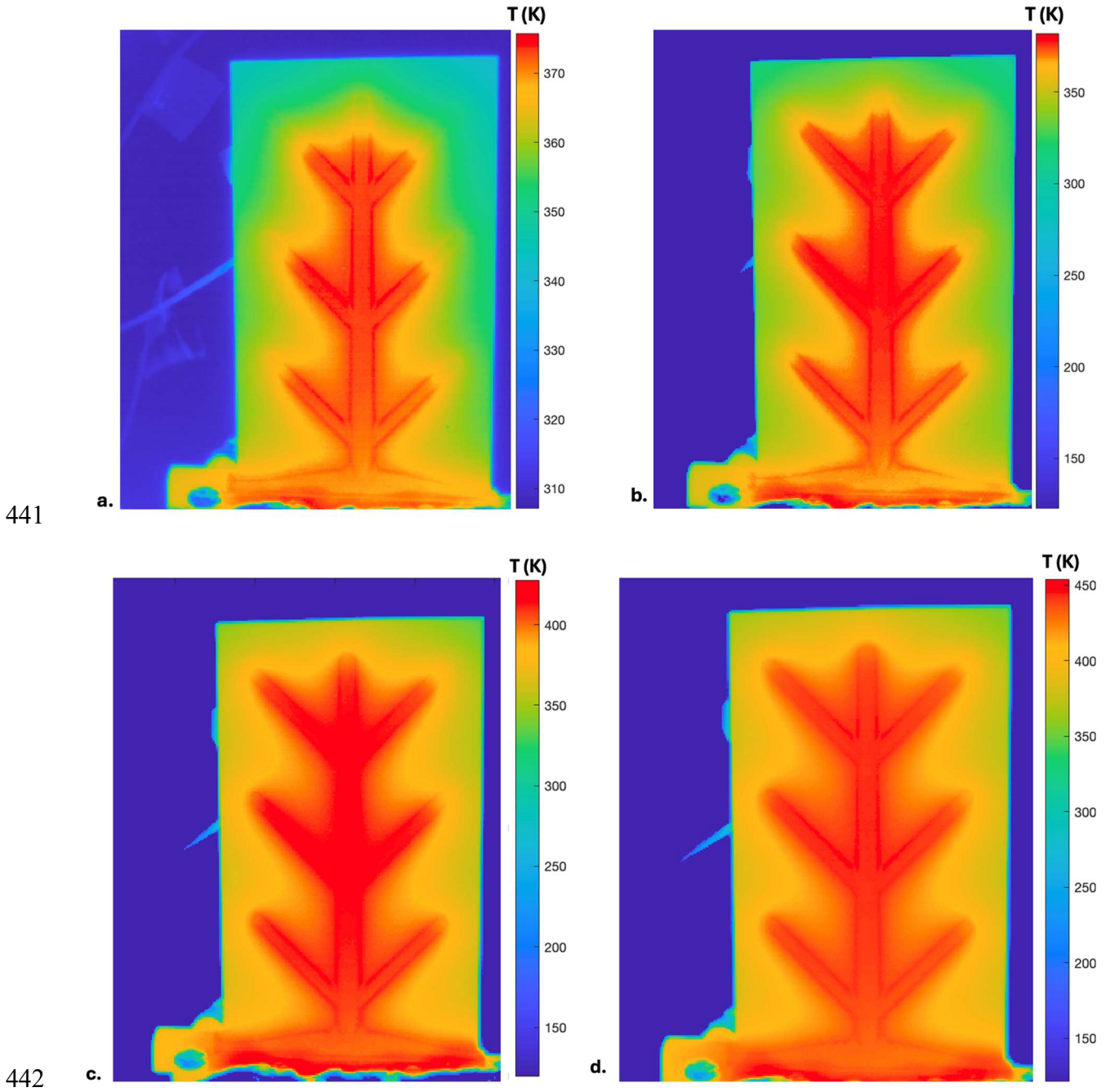
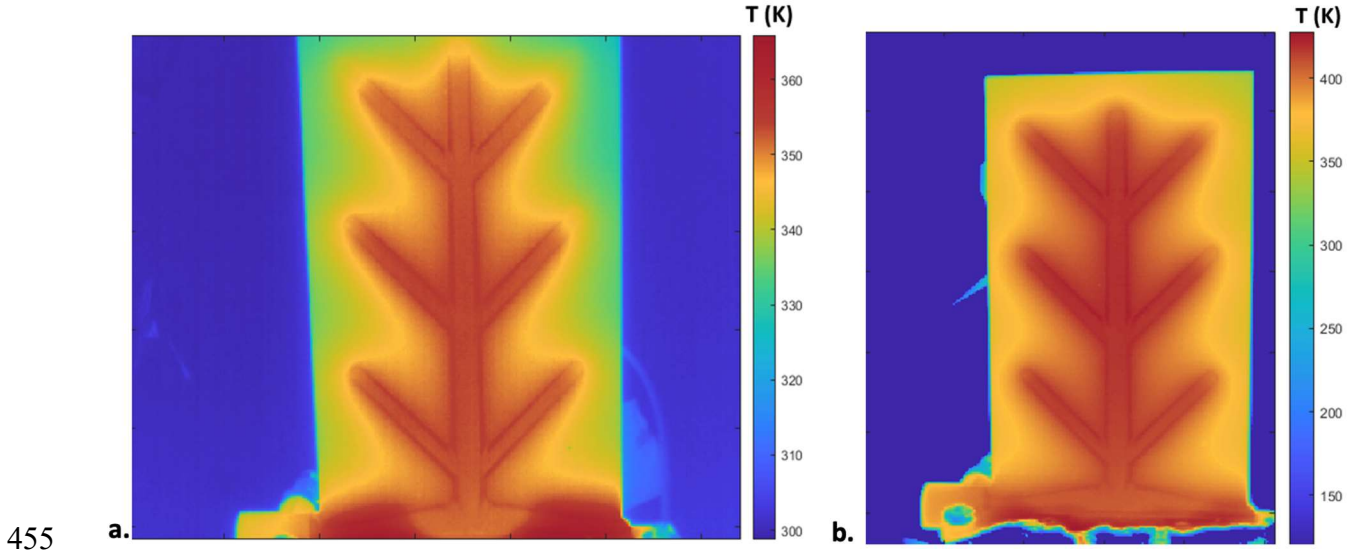


Figure 16: Thermal vacuum testing thermal images for titanium radiator prototype at:
a. $T_H = 100^\circ\text{C}$ (373.15 K), **b.** 150°C (423.15 K), **c.** 200°C (473.15 K), **d.** 235°C (508.15 K).

446 It was observed that the HPR performance at different temperatures was sensitive to fluid
 447 charge. Due to the nature of the charging process, the exact fluid quantity in the radiator was not
 448 quantitatively measured. A slightly greater liquid charge was needed to fully activate all heat pipe
 449 branches at high temperatures/heat input rates (Figure 16c-d). However, this same charge resulted

450 in degraded performance at lower heating rates (Figure 17a-b), with outer tips of heat pipe branches
 451 registering lower temperatures. We believe that excess liquid “floods” those heat pipe tips at lower
 452 heat input rates, rather than continuously circulating along the wick, as reported in previous similar
 453 cases (D. H. Lee & Bang, 2023). Reduced wick charge can fully activate heat pipe networks at
 454 lower temperatures but was found to cause high-temperature evaporator dry-out (Figure 17).



456 **Figure 17:** a. Temperature distribution at $T_H = 200^\circ\text{C}$, charge optimized for $T_H = 50^\circ\text{C}$, dry out
 457 at the evaporator, b. Temperature distribution at $T_H = 200^\circ\text{C}$, charge optimized for $T_H = 150^\circ\text{C}$,
 458 no dry out observed at the evaporator.

459

460 Overall radiator performance can be quantified by modeling the HPR as a gray diffuse emitter
 461 in a large isothermal environment (Equation 2). Here, Q_{heate} is the measured heater block input
 462 power (± 1 W), $Q_{loss,heater}$ is the radiated heat lost from the copper heater block and was estimated
 463 by running the heater in vacuum at the same temperature ranges as the radiator tests. ϵ_1 is the high
 464 temperature paint emissivity (0.94 ± 0.02) (García-Baños et al., 2022). σ is the Stefan-Boltzmann
 465 constant. A_s is the surface area of the radiator with an uncertainty of 5% (186 ± 14 cm²) due to the
 466 slight differences in dimensions between the actual CAD drawing and manufactured
 467 prototype. T_{avg} is the effective average HPR surface temperature (solved implicitly). At each
 468 operating temperature, T_∞ is an average of the values collected from 9 thermocouples distributed
 469 on the inside of the LN2 cooled sub-enclosure over the span of 5 minutes after stabilization. The
 470 heater power was averaged in a similar manner over the same time interval. Results are
 471 summarized in Table 5.

472

$$473 \quad Q_{rad} = q''_{rad} A_s = Q_{heater} - Q_{loss,heater} = \epsilon_1 \sigma A_s (T_{avg}^4 - T_\infty^4) \quad (2)$$

$$474 \quad Q_{loss,heater} = \epsilon_2 \sigma A_{S,copper\ block} (T_H^4 - T_\infty^4) \quad (3)$$

475
$$\eta_f = Q_{rad}/\varepsilon_1\sigma(T_H^4 - T_\infty^4) \quad (4)$$

476 **Table 5:** Summary of heat transfer measurements and results from thermal testing of prototype
 477 titanium heat pipe radiator.

T_H (K)	Q_{rad} (W)	q''_{rad} (W m ⁻²)	$Q_{loss,heater}$ (W)	T_∞ (K)	T_{avg} (K)	η_f (%)
376 ± 1	16.5 ± 1.1	880 ± 70	1.4 ± 0.5	145 ± 5	361 ± 8	85 ± 7
425 ± 1	25.9 ± 1.3	1380 ± 100	2.3 ± 0.8	139 ± 5	403 ± 7	80 ± 6
475 ± 1	35.2 ± 1.6	1880 ± 130	3.6 ± 1.3	140 ± 5	434 ± 8	70 ± 5
510 ± 1	44.9 ± 2.0	2390 ± 160	4.8 ± 1.7	140 ± 5	461 ± 8	67 ± 5

478
 479 Overall, these results indicate consistent performance of the HPR prototypes over a range of
 480 heat input temperatures up to 510 K. Fin efficiency does degrade with increasing temperature,
 481 which is consistent with conduction theory for radiating fins. However, overall surface efficiency
 482 maintained in the target range of $\eta_f \gtrsim 70\%$ for the tested conditions.

483
 484 **6. CONCLUSIONS**

485 This study details progress toward development of monolithic additively manufactured high
 486 temperature heat-pipe radiators (HPRs) with integral branching wicking heat pipes. Manufacturing
 487 strategies were developed to produce porous wicking structures in Inconel 718 and titanium alloys
 488 using laser powder bed fusion. Post-processing steps were developed to modify additively
 489 manufactured wick surfaces to be hydrophilic. Experiments were conducted to measure capillary
 490 properties of wicks, including permeability (K), porosity (ϵ), and average pore radius (r_{pore}).
 491 These tests identified wick fabrication strategies that yield high performance metrics (K and
 492 K/r_{pore}), compared with conventionally manufactured wicks. These wick parameters and post-
 493 processing steps were applied to produce full CP-Ti HPR prototypes. A prototype HPR was tested
 494 in a relevant cryogenic thermal environment. With the appropriate fluid charge, the specimen
 495 achieved nearly isothermal heat pipe operation and $\eta_f \gtrsim 67\%$ for heat input at T_H up to 510 K
 496 with an areal density of ~ 2.13 kg m⁻². This aligns with objectives proposed by NASA.

497 In upcoming work, the team is planning to evaluate the AM branching HPR concept for larger
 498 prototypes to assess scalability. Modeling studies are underway to optimize layout and geometry
 499 of branching heat pipe networks for larger HPRs. Further work is planned to integrate coupling
 500 fluid loop heat transfer in evaporator sections and assess practical factors, such as micrometeoroid
 501 and orbital debris impact risks and deployment strategies for large HPR arrays.

502 **ACKNOWLEDGMENTS**

503 The authors wish to acknowledge generous support from the U.S. National Aeronautics and Space
504 Agency Space Technology Mission Directorate – Early Stage Innovations (Grant
505 80NSSC22K0260).

506

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