



Multilayer Insulation for In-Space Cryogenic Applications

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Logistics



- **Egress**
- **Restrooms**
- **Breaks**



Outline



- **Cryogenic MLI Introduction**
 - Heat Transfer basics
 - Terminology
 - Fundamental
 - History
- **Cryogenic MLI blanket variables**
 - Empirical Equations
 - Vacuum Pressure
 - Layer Count
 - Layer Density
 - Warm Temperature
 - Cold Temperature
 - Transient Performance
- **Cryogenic MLI materials**
 - Reflectors
 - Perforations
 - Spacers
- **Cryogenic MLI builds**
 - Seaming
 - Venting
 - Small Penetrations
 - Med Penetrations
 - Grounding
 - Repeatability
- **IMLI**





About the Instructor



- **Cryogenics Test Laboratory – KSC (10 yrs)**
- **Fluid and Cryogenics Group (LTF) – GRC (10 yrs)**
- **Over twenty years hands on experience with liquid nitrogen and other cryogenic fluids**
- **Past Experience:**
 - Multiple liquid hydrogen tests
 - Structural Heat Intercept, Insulation, and Vibration Evaluation Rig
 - Integrated Refrigeration and Storage Oxygen and Hydrogen Demonstrations
 - 9 years of insulation thermal performance testing & test design
 - Space Shuttle Return to Flight
 - Trouble shooting & improvement of operations at KSC launch pads
 - 4 patents (3 from insulation measurement devices)
 - Operation of nearly 10 different calorimeters varying in size
 - Much of the data you will see today originates from testing and analysis I was directly involved in



Course Objective



- **A fundamental understanding of what multilayer insulation is.**
- **Understand the history and development of cryogenic multilayer insulation.**
- **A fundamental understanding of the different elements associated with cryogenic multilayer insulation design, fabrication, and installation.**
- **A list of places to go for more information.**
- **To touch the inquisitive nature of an engineer.**



Introduction to Multilayer Insulation





What is Multilayer Insulation?

- **Multilayer Insulation targets the reduction of all types of heat transfer:**
- **Gas Convection/Gas conduction:**
 - On the launch pad, closely spaced layers significantly impede gas convection to where a MLI blanket can loosely be modeled as a stagnant gas pocket.
 - In a vacuum: Either through a man-made vacuum or launching to space, the gas between the layers is removed, thereby, eliminating any convective heat transfer leaving only free molecular flow of gas particles between layers.
- **Radiation:**
 - The outer layers of a multilayer insulation blankets are devised to allow the minimal amount of heat into the system from a radiative source.
 - The inner layers of a multilayer insulation blanket are high reflectivity, low absorptivity materials that allow on the order of less than 5% of energy to be transferred radiatively between individual layers.
- **Solid Conduction:**
 - Ideal multilayer insulation systems have no materials / contact between layers (i.e. floating shields)
 - Low conductivity spacer materials (dacron/nylon, paper, other polymers) are usually [but not always] used in between layers to minimize what is needed to support blankets.
 - Low constrictive pressure (large spaces between layers; low layer density) forcing contact between spacer materials and reflector materials.



How was Multilayer Insulation developed?

- **Sir James Dewar (c 1900)**
 - Double wall glass container with vacuum in between the two walls
 - Silvering of inner wall to lower radiation heat load and keep cryogenics around longer
- **Peterson (c 1957)**
 - Multiple reflective layers stacked to improve the performance and reduce radiative heat loads
- **Era of AD Little Company (Black, Glaser, etc.; 1960s)**
 - Multiple test setups
 - Testing all different types of reflectors and spacers
 - Begin the area of more detailed characterization
- **Era of Primes (Lockheed, General Dynamics; 1970s)**
 - Characterization of specific MLI designs favored by principle aerospace primes
 - Ready for implementation on an array of applications
 - Development of basic spacecraft MLI design techniques
 - Industrial uses baselined for earth-based uses
- **Era of Government (NASA; 1980s – 2010s)**
 - Maintain testing capabilities and understanding of system performance
 - Development of new philosophies (variable density, spray foam / GN2 purging)
 - Implementation of prime specific designs on many [relatively] small orbital observatories/dewars
- **Era of Primes Part 2 (2020s?)**
 - Required implementation to meet ambitious architectural goals
 - Sharing of knowledge retained by government back to primes



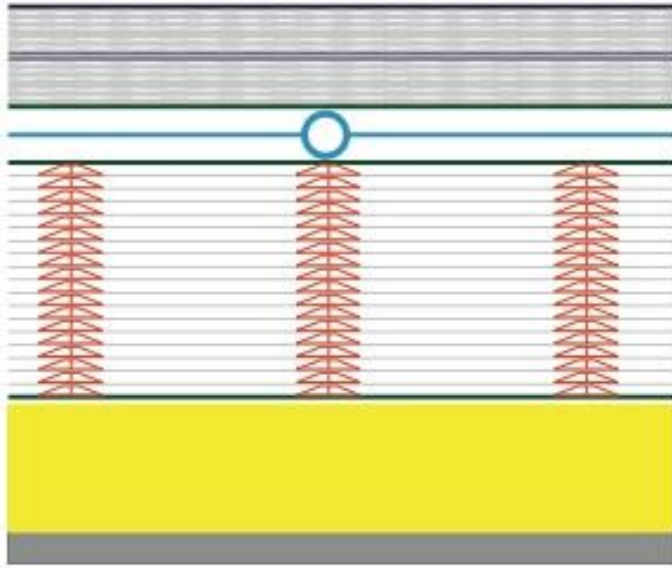
Multilayer Insulation Basics - Terminology



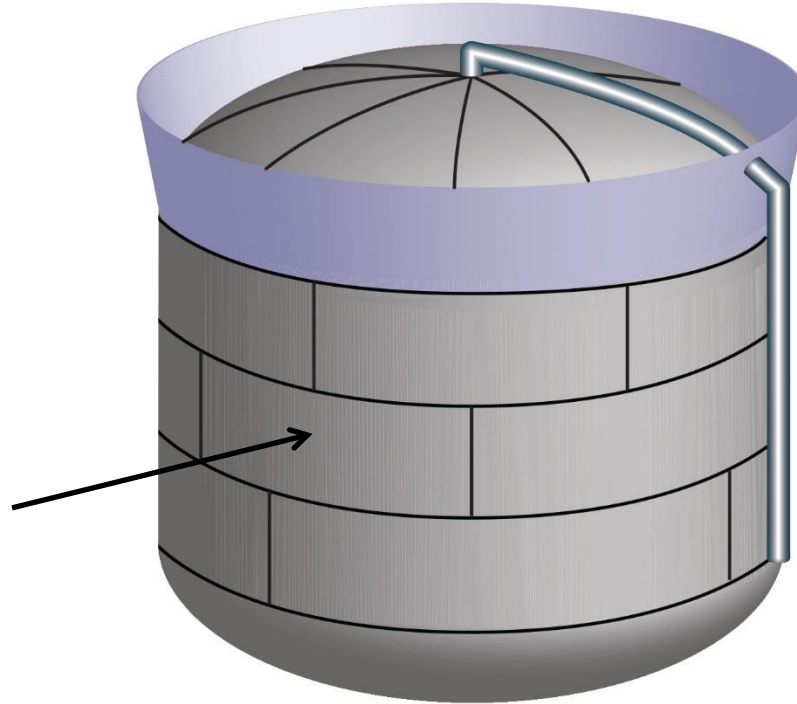
- **Multilayer Insulation and Super Insulation (what it was called prior to the mid-60s) are generally the same thing.**
- **Multilayer Insulation = MLI**

- **Warm Boundary Temperature (WBT)**
 - Can be either the outer layer temperature of the MLI blanket or the radiative source temperature of a test.
 - Typically the two are fairly close
- **Cold Boundary Temperature (CBT)**
 - Typically the temperature of the cryogenic fluid that the MLI is protecting
 - Generally referred to in a generic sense (i.e. LH_2 NBP = 20.4 K, often referred to at 20 K)
- **Cold Vacuum Pressure (CVP)**
 - The vacuum pressure achieved when a test article is full of cryogenics or at appropriate cold boundary temperature.
- **Warm Vacuum Pressure (WVP)**
 - The vacuum pressure achieved via mechanical pumping only (system still warm)

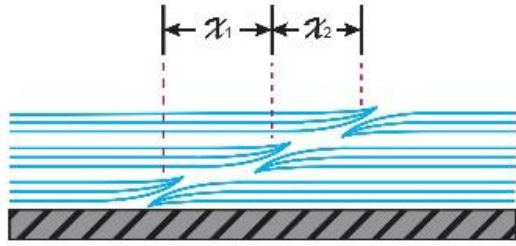
Improved Fundamental Understanding of Super Insulation (IFUSI)



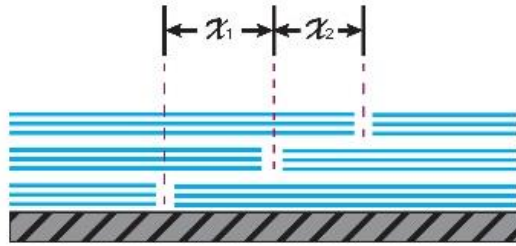
- MLI Blankets
- Traditional
 - IMLI
 - Hybrid



Improved Fundamental Understanding of Super Insulation (IFUSI)



3 Blanket Overlap Staggered

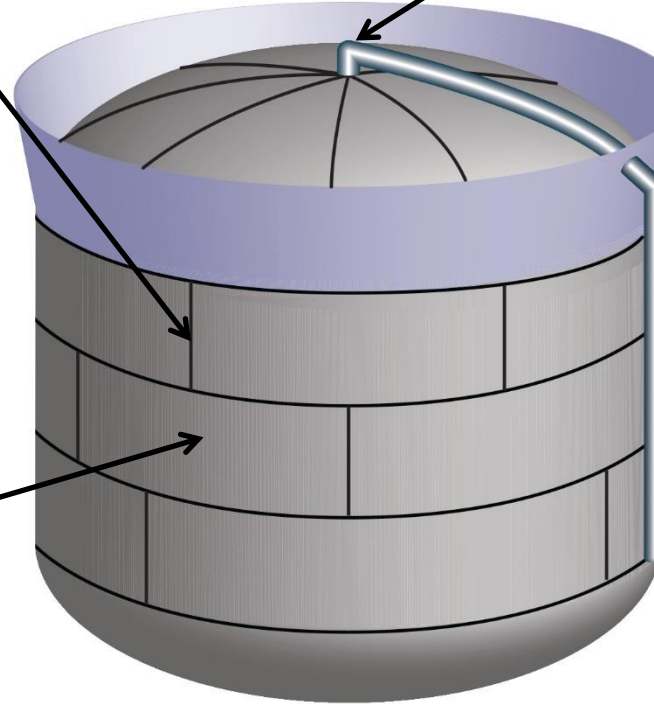


3 Blanket Staggered

Seams

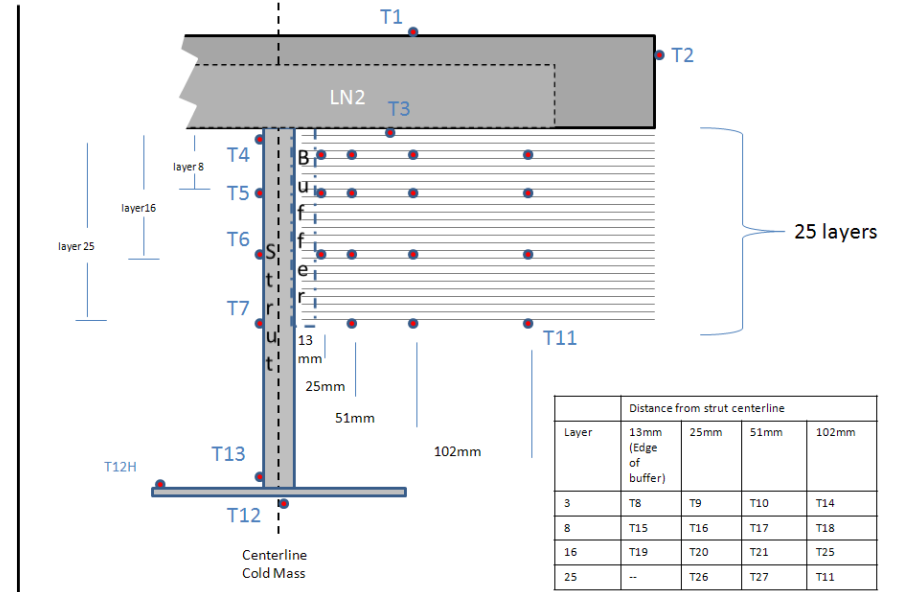
MLI Blankets

- Traditional
- SS-MLI
- IMLI
- Hybrid

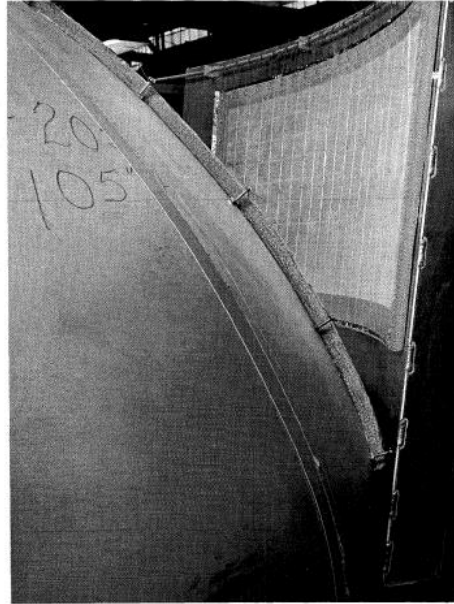


Penetration Integration:

- NASA-TP-2012-216315



Improved Fundamental Understanding of Super Insulation (IFUSI)



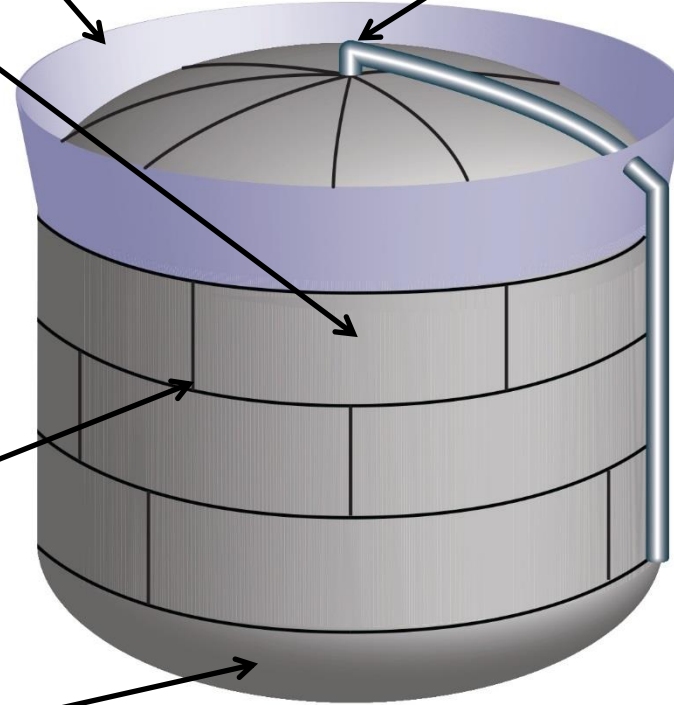
Skirt Integration

Seams

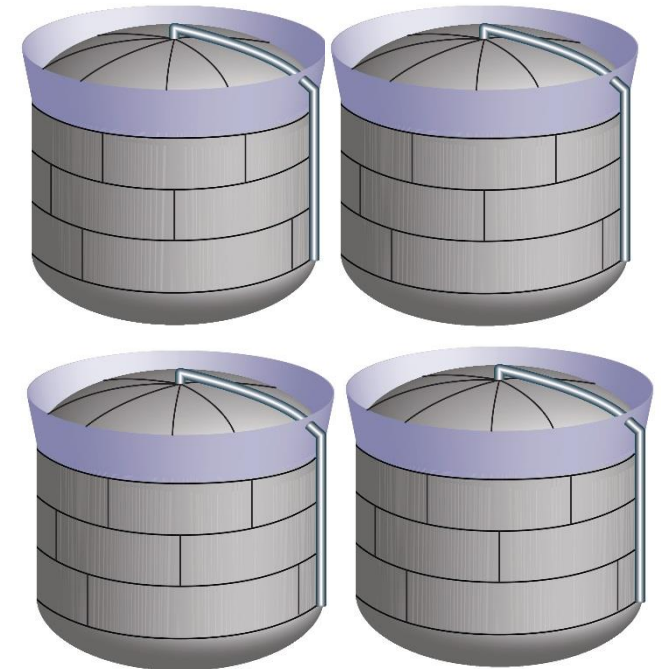
Penetration Integration:
- NASA-TP-2012-216315

MLI Blankets
- Traditional
- SS-MLI
- Hybrid

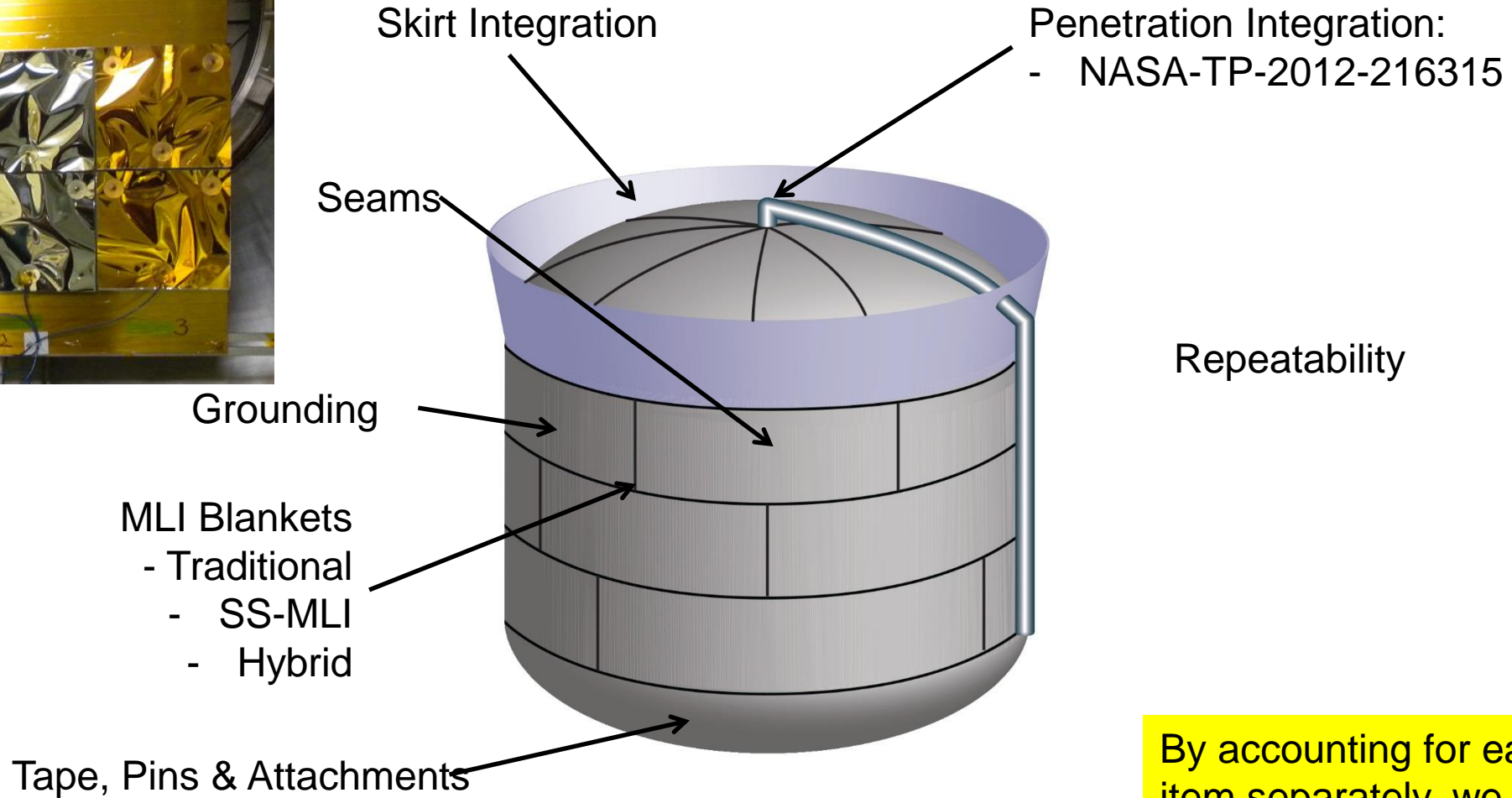
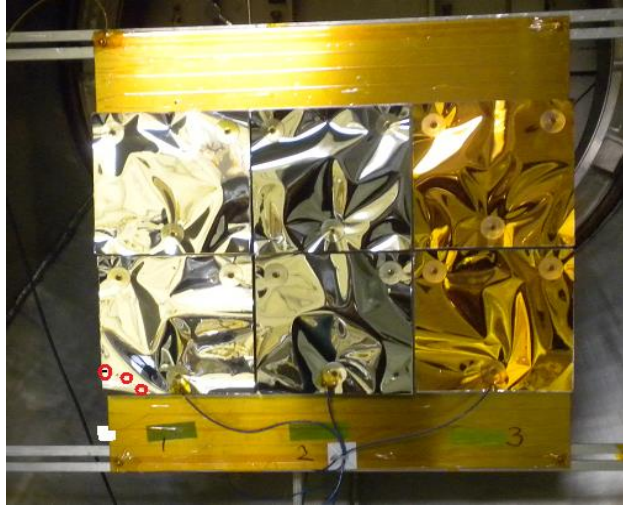
Tape, Pins & Attachments



Repeatability



Improved Fundamental Understanding of Super Insulation (IFUSI)



By accounting for each item separately, we can more accurately predict total MLI performance.



Basic MLI Fundamentals



- **System model must match the hardware design**
 - Include the different aspects of implementation discussed in previous slides
 - System-level analysis must be setup account for inefficiencies
 - A majority of current tools are not set up to do this well
 - Hardware implementation and components used must reflect what is in system-level analysis
- **Must understand environments**
 - Thermal
 - Vibration / Acoustic
 - Electrical
 - Launch Pad / Evacuation
 - Ultimate vacuum pressure
- **Cryogenic MLI and Spacecraft MLI are fundamentally different**
 - Cryogenic MLI requires at least one order of magnitude better system level performance.
 - Spacecraft MLI has a fairly constant “Warm Boundary Temperature” whereas Cryogenic MLI has fairly constant “Cold Boundary Temperatures”.
 - Cryogenic MLI requires more careful treatment in “off-nominal” environments (pad, etc).



Which MLI System is Better?



	<u>Sys 1</u>	<u>Sys 2</u>
$Q (W)$	0.222	0.224
$q'' \left(\frac{W}{m^2} \right)$	0.697	0.605
$k \left(\frac{mW}{m \cdot K} \right)$	0.026	0.104



MLI Blanket Variables





Empirical Equations – Lockheed & McDonnell Douglas

In the early 1970s, NASA awarded contracts to various companies to develop equations for MLI:

Examples from Lockheed/Keller:

Unperforated Shields:

$$q_T = \frac{C_s (\bar{N})^{2.56} T_m (T_H - T_C)}{N_s} + \frac{C_r \epsilon_{TR} (T_H^{4.67} - T_C^{4.67})}{N_s} \quad (4-56)$$

$$+ \left\{ \begin{array}{l} \frac{C_g P}{N_s} (T_H^{0.52} - T_C^{0.52}) \text{ for } \text{GN}_2, \text{ or} \\ \frac{C_g P}{N_s} (T_H^{0.26} - T_C^{0.26}) \text{ for } \text{GHe} \end{array} \right.$$

where: $C_s = 8.95 \times 10^{-8}$, $C_r = 5.39 \times 10^{-10}$, $C_g(\text{GN}_2) = 1.46 \times 10^4$, and
 $C_g(\text{GHe}) = 4.89 \times 10^4$ for \bar{N} in layers/cm, T in $^{\circ}\text{K}$, P in torr,
 and q in W/m^2

or: $C_s = 8.06 \times 10^{-10}$, $C_r = 1.10 \times 10^{-11}$, $C_g(\text{GN}_2) = 3.44 \times 10^3$, and
 $C_g(\text{GHe}) = 1.33 \times 10^4$ for \bar{N} in layers/in., T in $^{\circ}\text{R}$, P in torr,
 and q in Btu/hr ft^2

and: $\epsilon_{TR} = 0.031$

Th	293 K	
Tc	78 K	
N	30 layers	
Nbar	14.1 layers/cm	
P	2.00E-06 Torr	

NASA Report #		Heat Flux
TM-2004-213175	Modified Lockheed Equation - CDMLI	0.411 W/m ²
NAS3-14377	Lockheed Unperforated Mylar & Silk Net	0.294 W/m ²
NAS3-14377	Lockheed Perforated S-604	0.350 W/m ²
NAS3-14377	Lockheed Perforated S-603	0.321 W/m ²
NAS3-14377	Lockheed Perforated S-602	0.280 W/m ²
NAS3-14377	Lockheed Perforated 937	0.346 W/m ²
NAS3-14377	Lockheed Perforated 937S	0.321 W/m ²
NAS3-14377	Lockheed As received Silk (non-perf)	0.227 W/m ²
NAS8-20758	Lockheed DAM/Nylon	2.681 W/m ²
NAS8-21400	Fredrickson	5.813 W/m ²
TM-2004-213175	MHTB Style - VDMLI	0.298 W/m ²

Examples from Douglas/Fredrickson:

DAM/Nylon Net:

$$K_e = 3.02 \times 10^{-15} \bar{N}^{3.4} T_m + 2.21 \frac{\sigma(T_h^2 + T_c^2) (T_h + T_c) t}{(N-1) \left(\frac{1}{\epsilon_1} + \frac{1}{\epsilon_2} - 1 \right)}$$

$$\epsilon_1 = 0.03, \quad \epsilon_2 = 0.03$$



Modified Lockheed Forms



- In concert with the development and testing of variable density multilayer insulation on the Multipurpose Hydrogen Test Bed (MHTB) at MSFC, Hastings and Hedayat developed what they called a “modified Lockheed equation”[1].

Dacron conduction from McIntosh [2]

$$q_{\text{total}} = 2.4 \times 10^{-4} * (0.017 + 7 \times 10^{-6} (800 - T) + 0.0228 \ln(T)) * (N^*)^{2.63} (T_H - T_C) / N_s$$

$$+ 4.944 \times 10^{-10} \epsilon (T_H^{4.67} - T_C^{4.67}) / N_s + 1.46 \times 10^4 P (T_H^{0.52} - T_C^{0.52}) / N_s \quad (13)$$

Radiation with 2" dia perfs Normal Lockheed gas conduction

- Johnson [3] developed a “New-Q” equation based on experimental data correlations using Dacron conduction term and unperforated Lockheed double aluminized mylar terms.

$$\frac{Q}{A} = \frac{\left(2.4E - 4 * \left(0.017 + 7E - 6 * (800 - T_{\text{avg}}) + 2.28e - 2 * \ln(T_{\text{avg}}) \right) \right) \bar{N}^{2.63} (T_h - T_c)}{N_s + 1}$$

$$+ \frac{5.39E - 10 * \epsilon * (T_h^{4.67} - T_c^{4.67})}{N_s} + \frac{1.46E4 * P * (T_h^{0.52} - T_c^{0.52})}{N_s}$$

- L. Hastings, A. Hedayat, et al., Analytical Modeling and Test Correlation of Variable Density Multilayer Insulation for Cryogenic Storage, NASA-TM-2004-213175, 2004.
- G. E. McIntosh, Layer by Layer MLI Calculation using a Separated Mode Equation, in: Advances in Cryogenic Engineering, Vol 39B, Plenum Press, NY, 1993, pp. 1683-1690.
- Johnson, W.L. and Fesmire, J.E., “Thermal Performance of Low Layer Density Multilayer Insulation Using Liquid Nitrogen”, *Advances in Cryogenic Engineering*, Vol. 57A, American Institute of Physics, Melville, NY, 2012. Pg. 39-46.



Analytical Look at Cryogenic Modeling Approaches

Lockheed style:

Warm Boundary Temperature (T_h)
Cold Boundary Temperature (T_c)

Gas Pressure (P)
-note only good less than 10^{-4} Torr

Layer Density (\bar{N}) Room Temp Emissivity (ϵ)

$$q'' = \frac{C_s * \bar{N}^{2.63} * (T_h - T_c) * (T_h + T_c)}{2 * (N + 1)} + \frac{C_R * \epsilon * (T_h^{4.67} - T_c^{4.67})}{N} + \frac{C_G * P * (T_h^{0.52} - T_c^{0.52})}{N}$$

Number of layers (N)

Empirical Coefficients

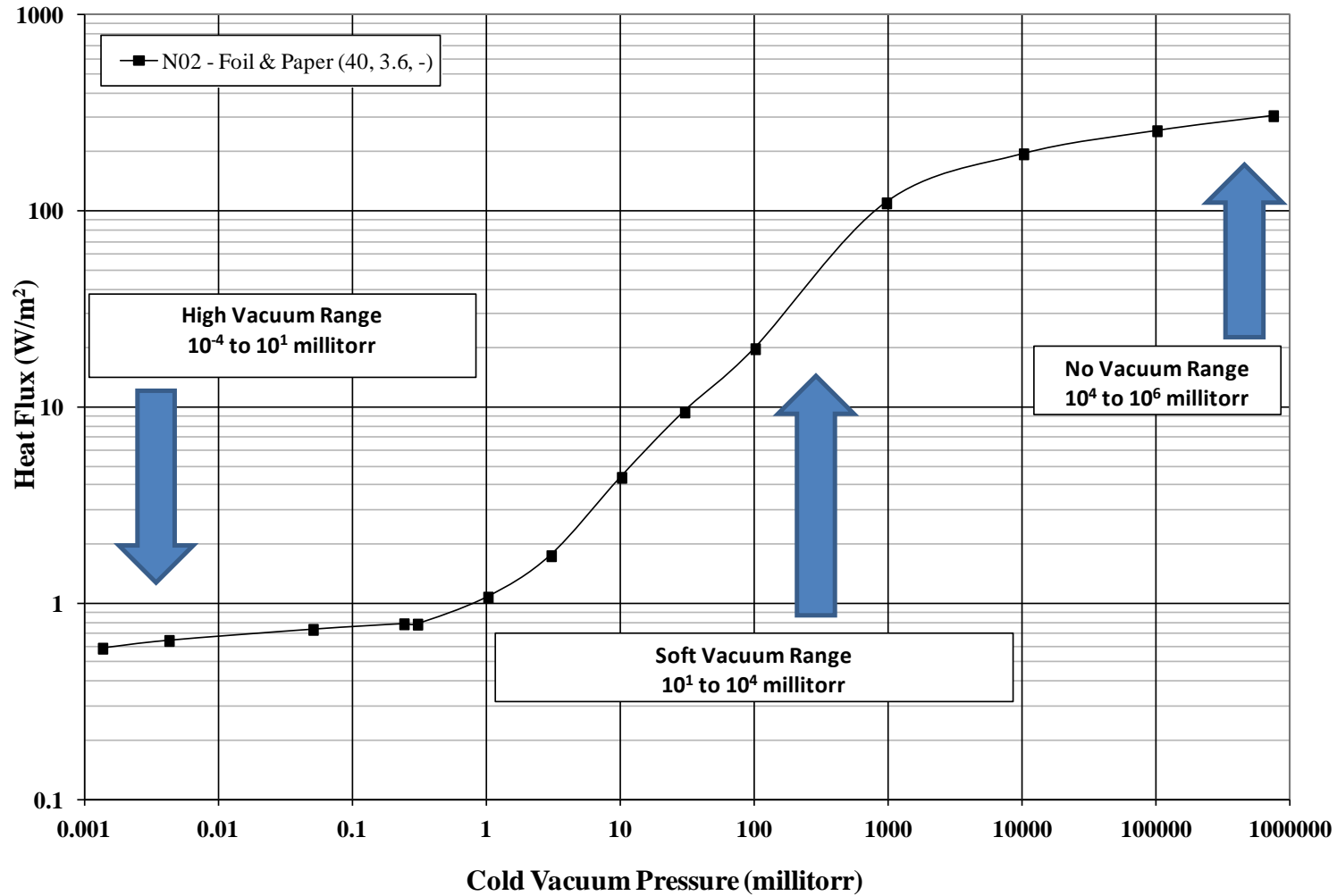
McIntosh / Layer by Layer style:

$$q'' = \frac{\sigma (T_h^4 - T_c^4)}{\left(\frac{1}{\epsilon_h} + \frac{1}{\epsilon_c} - 1\right)} + C_1 P \alpha (T_h - T_c) + \frac{C_2 f k (T_h - T_c)}{\Delta x}$$

Measurable Values



The Effect of Vacuum Pressure on MLI

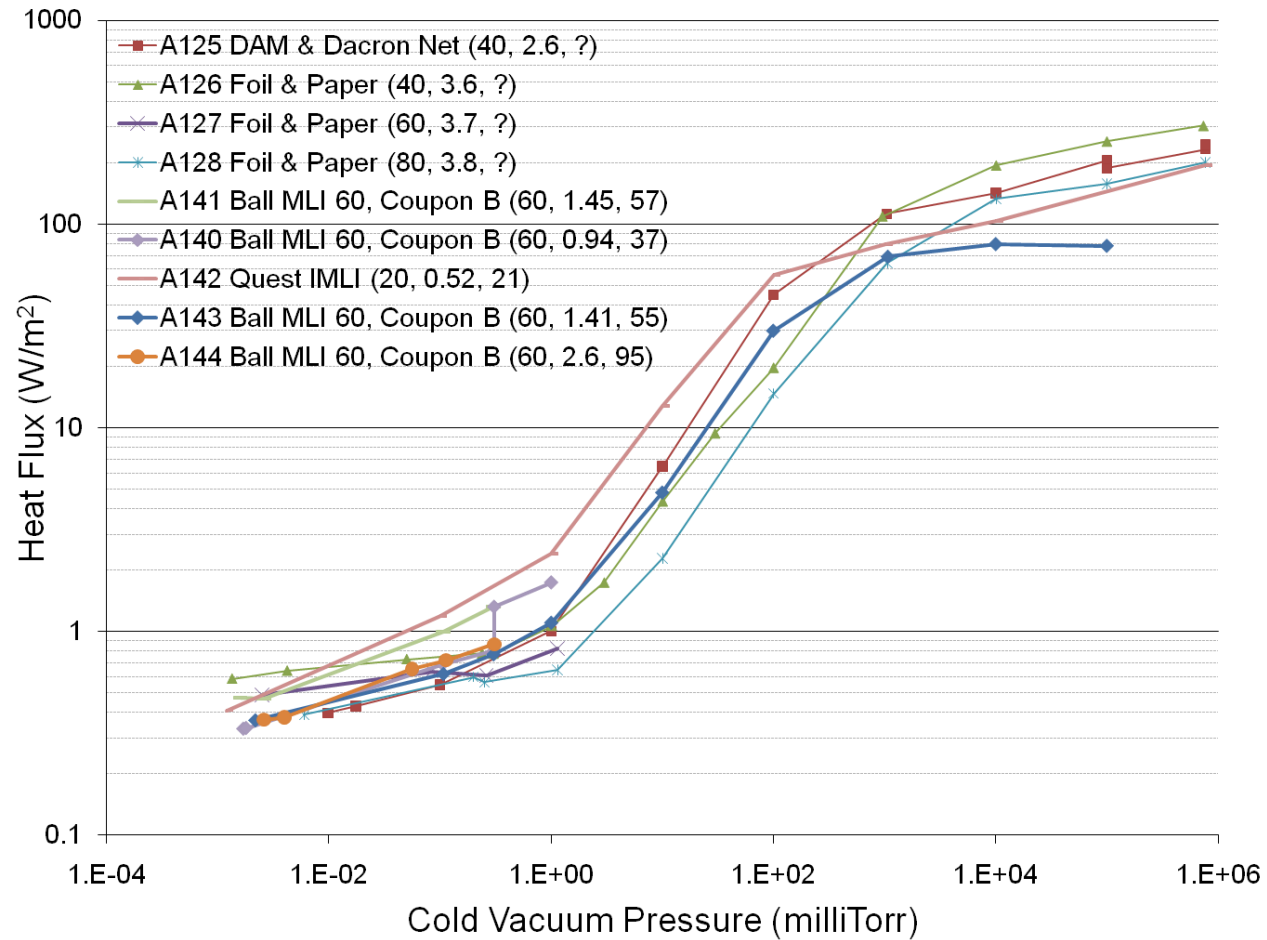




Degraded (Soft) Vacuum



Why is there so much data spread at High Vacuum and Soft Vacuum?



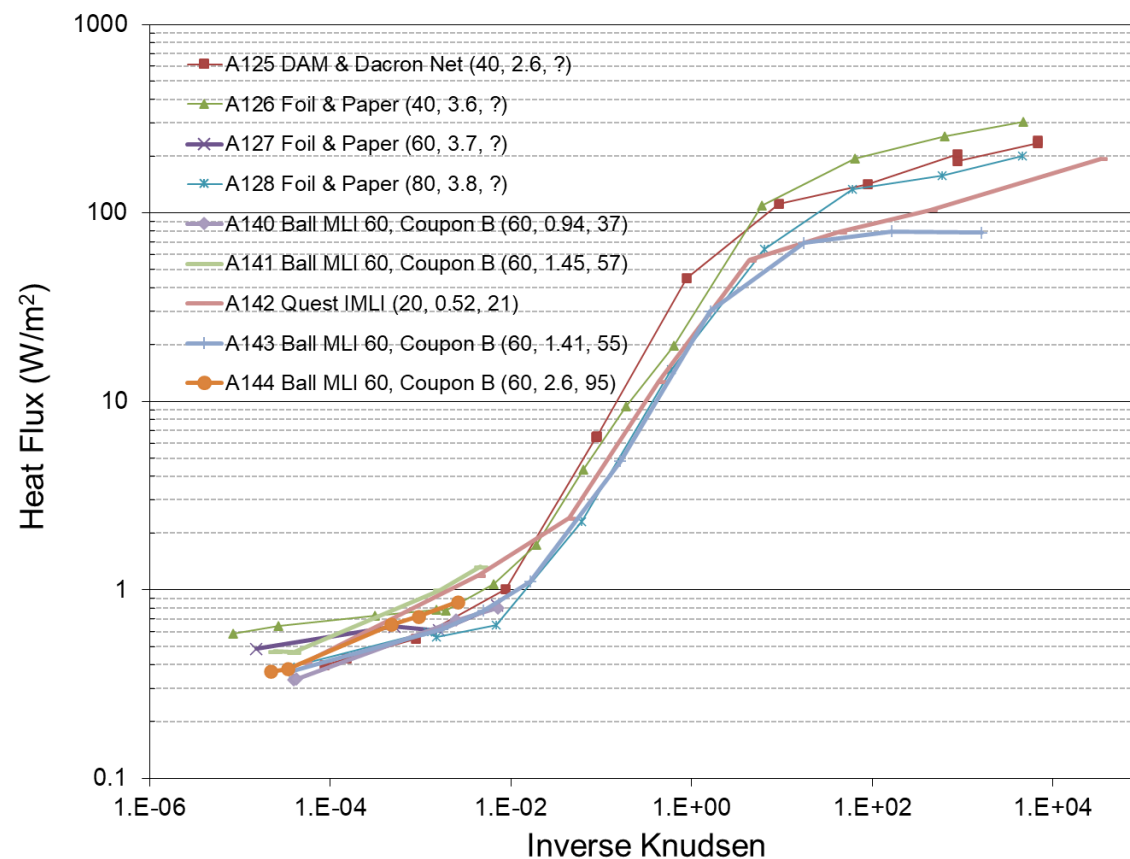


Knudsen Number

- Historical MLI performance predicted by vacuum pressure
- Here we plot against the mean inverse Knudsen number

$$\lambda = \frac{k_B T}{\xi^2 P} \quad Kn = \frac{\lambda}{N/\bar{N}} \quad iKn = \frac{N}{\bar{N}\lambda}$$

- Assume same pressure through blanket as chamber
- Mean temperature
- ξ is the diameter of the gas molecule, nitrogen: 3.14E-10m





Boundary Temperatures

$$q'' = \frac{C_s * \bar{N}^{2.63} (T_h - T_c) * (T_h + T_c)}{2 * (N + 1)} + \frac{C_R * \epsilon * (T_h^{4.67} - T_c^{4.67})}{N} + \frac{C_G * P * (T_h^{0.52} - T_c^{0.52})}{N}$$

Based on this equation format, which boundary temperature is more important?

Warm Boundary Temperature (K)

Cold Boundary Temperature (K)

Heat Flux (W/m ²)	345	290	240	200
100	0.54	0.28	0.14	0.07
77	0.55	0.28	0.15	0.08
20	0.56	0.30	0.16	0.09
4	0.56	0.30	0.16	0.09

Approximately factors of 2 in heat flux.

Next temp in sequence is 165 K.

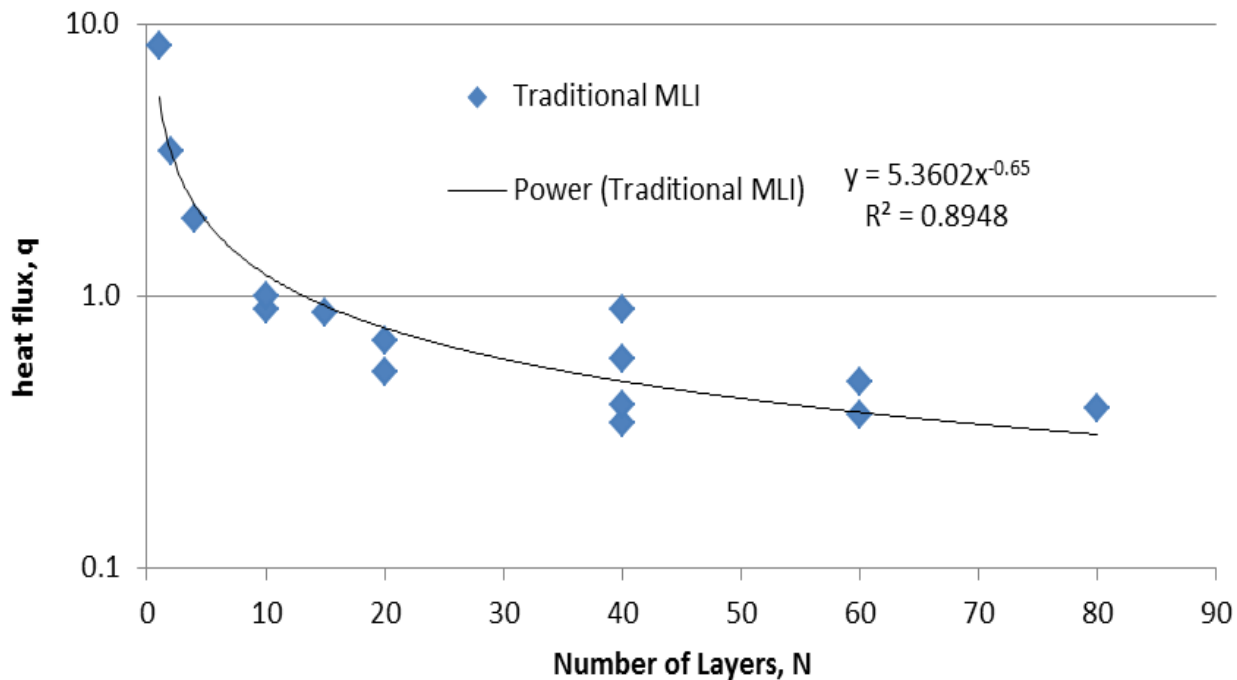
4% -----> 24% change

Data generated using the above equation for 30 layers at 14.1 layer/cm and high vacuum

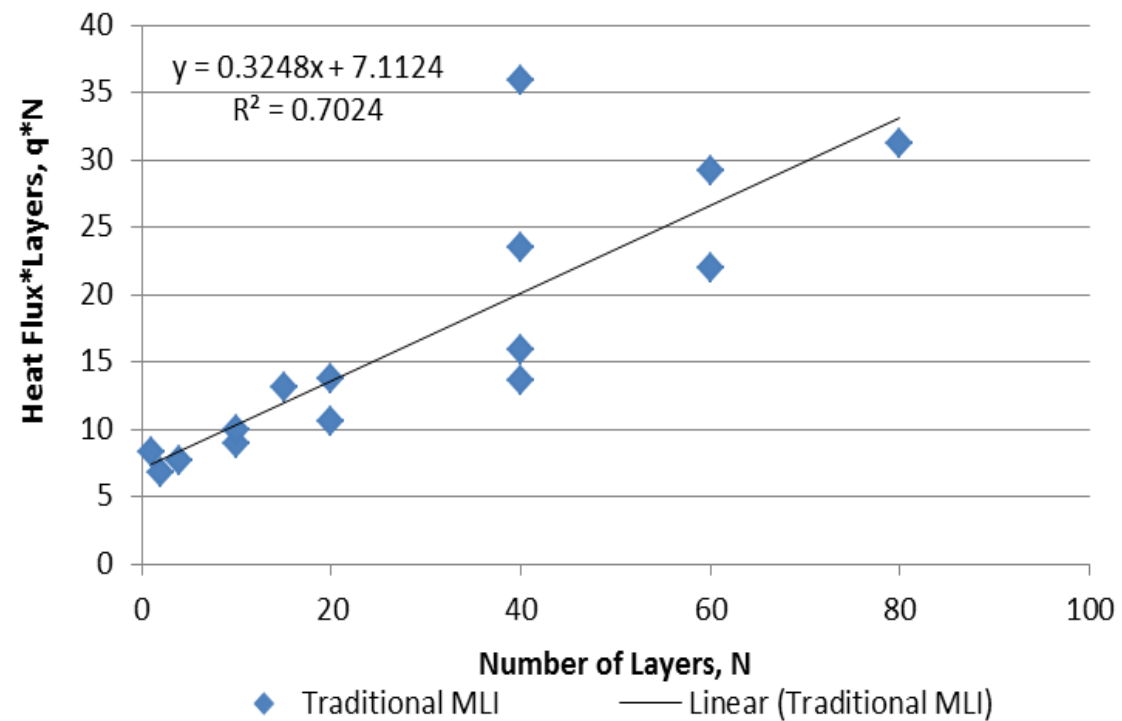


Number of Layers

Variation of heat flux q with the number of layers



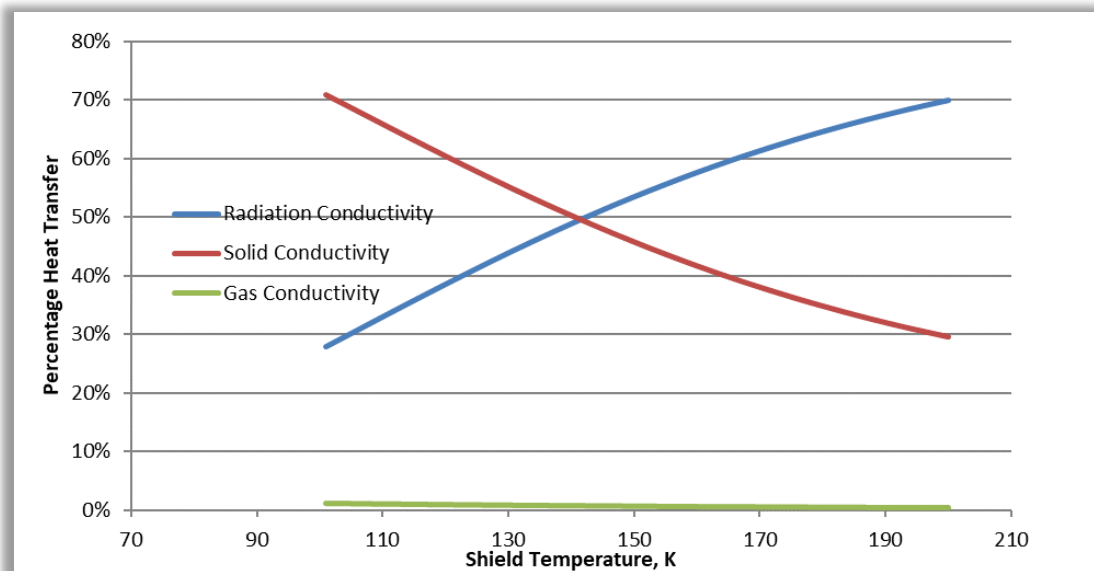
Variation of the quantity $q*N$ with the number of layers





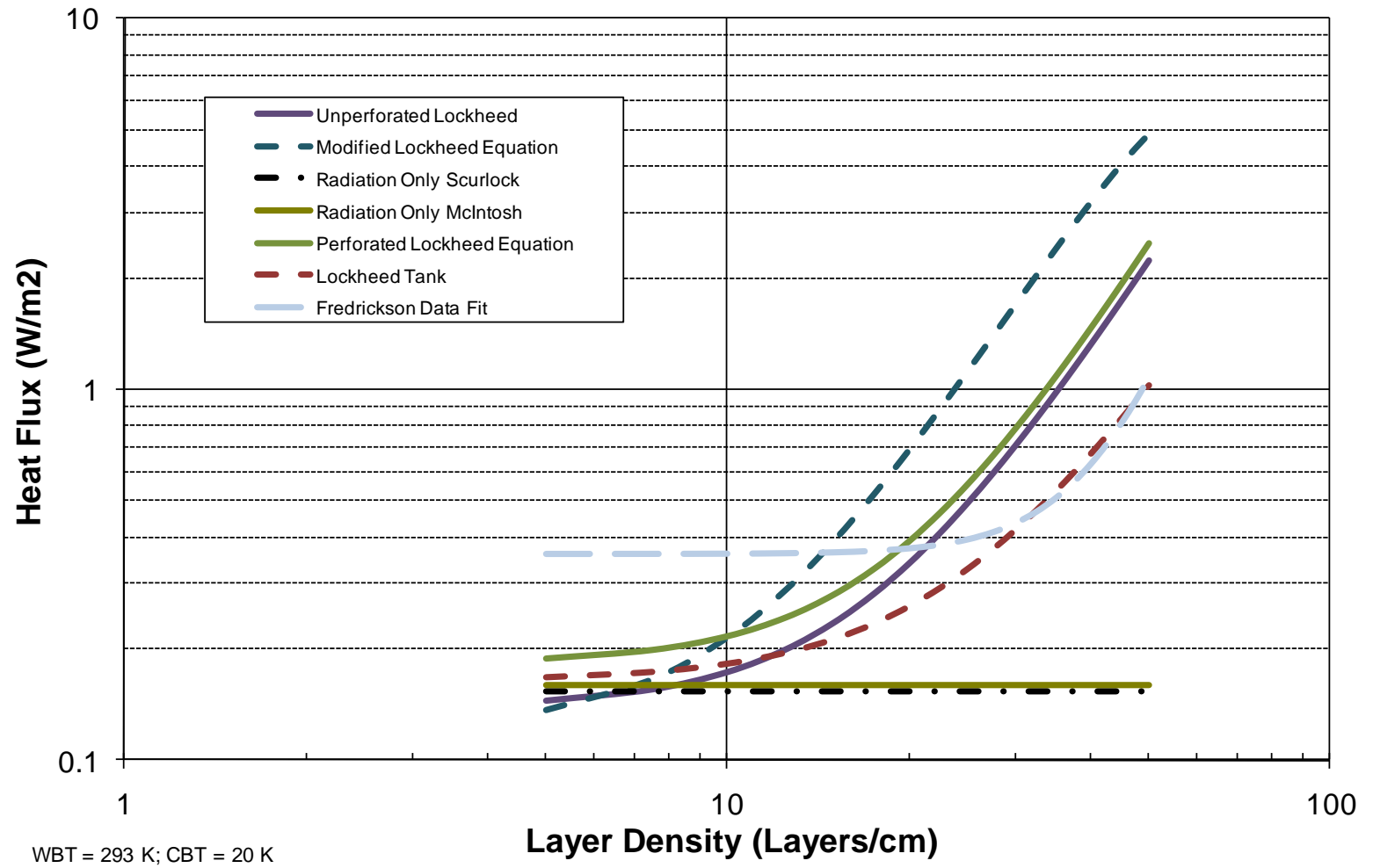
Conduction vs Radiation Heat Transfer

- **Given different situations, either solid conduction or radiation heat transfer could be the dominant form of heat transfer in an MLI system.**
- **The plot below is an example of what the relative heat transfer percentages might be through a blanket.**
 - In this example, the warm portion of the blanket is radiation dominant while the cold portion is solid conduction dominant.
 - Gas conduction plays a very small role in heat transfer at a high vacuum MLI system.





Effect of Layer Density at a Constant Number of Layers

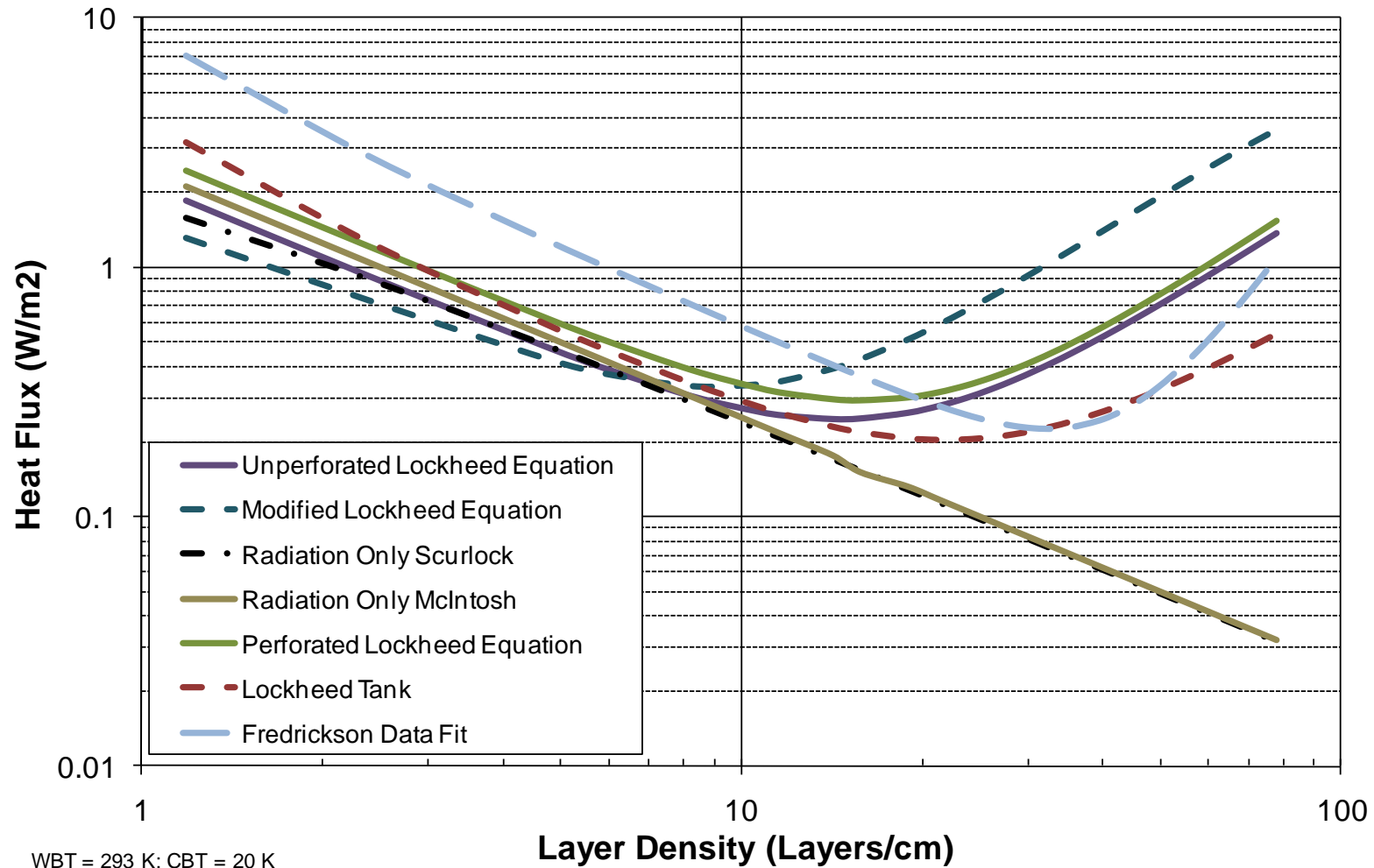


WBT = 293 K; CBT = 20 K
Constant number of layers: 40

Johnson, W.L., "Optimization of Layer Densities for Multilayered Insulation Systems," *Advances in Cryogenic Engineering*, Vol. 55A, American Institute of Physics, Melville, NY, 2010. Pg. 804-811.



Layer Density at a Constant Thickness

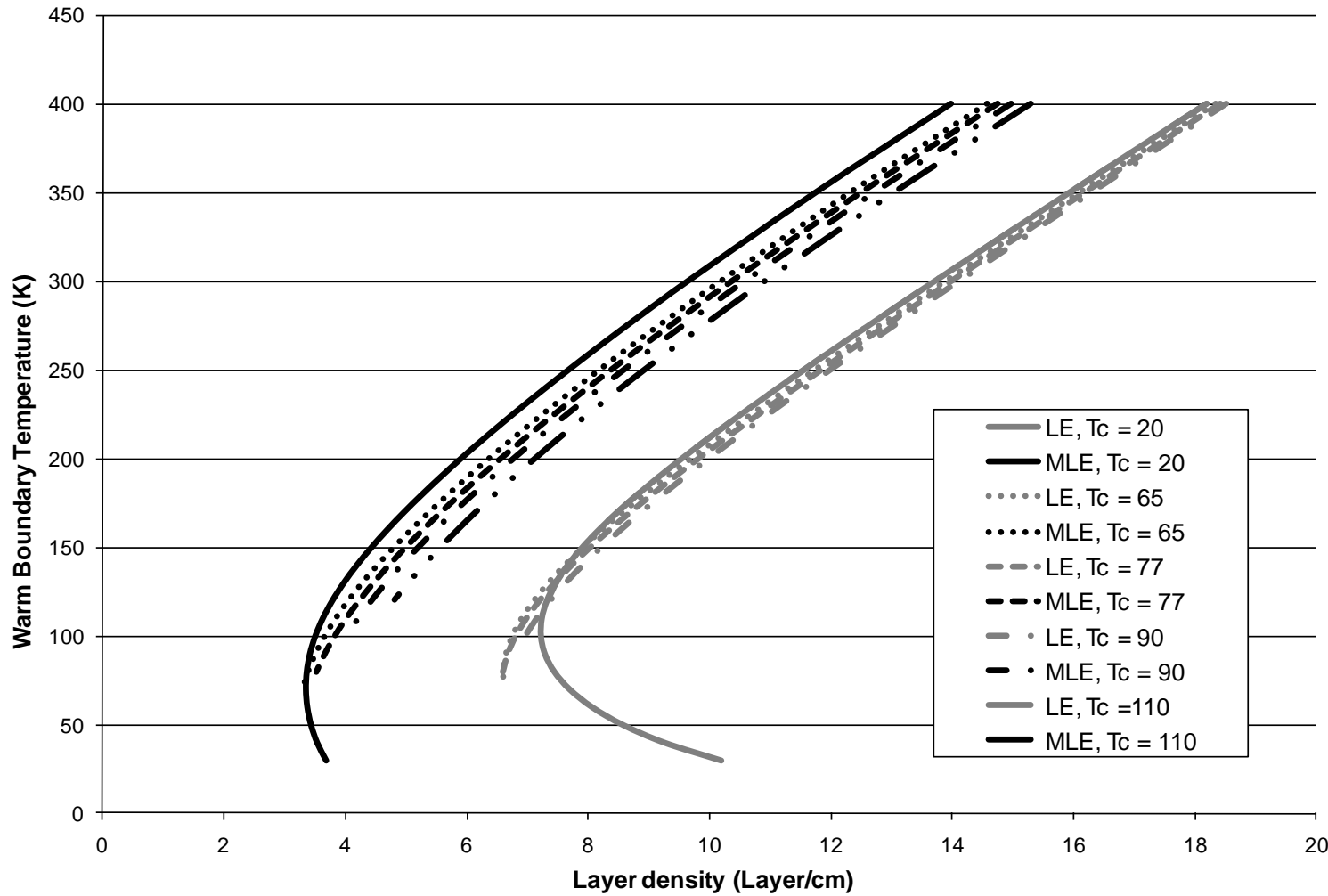


WBT = 293 K; CBT = 20 K
Constant thickness of 2.5 cm

Johnson, W.L., "Optimization of Layer Densities for Multilayered Insulation Systems," *Advances in Cryogenic Engineering*, Vol. 55A, American Institute of Physics, Melville, NY, 2010. Pg. 804-811.



Optimal Layer Densities



LE is the Lockheed Unperforated equation [equation 4-14]
 MLE is the Modified Lockheed Equation [equation 13]

$$\bar{N}_{opt,MLE} = \left[\frac{C_R * \epsilon * (T_h^{4.67} - T_c^{4.67}) + C_G * P * (T_h^{0.52} - T_c^{0.52})}{1.63 C_s * (T_h - T_c)} \right]^{\frac{1}{2.63}}$$

$$\bar{N}_{opt} = \left[\frac{C_R * \epsilon * (T_h^{4.67} - T_c^{4.67}) + C_G * P * (T_h^{0.52} - T_c^{0.52})}{0.78 C_s * (T_h^2 - T_c^2)} \right]^{\frac{1}{2.56}}$$

Johnson, W.L., "Optimization of Layer Densities for Multilayered Insulation Systems," *Advances in Cryogenic Engineering*, Vol. 55A, American Institute of Physics, Melville, NY, 2010. Pg. 804-811.



Non-Constant MLI Layer Densities

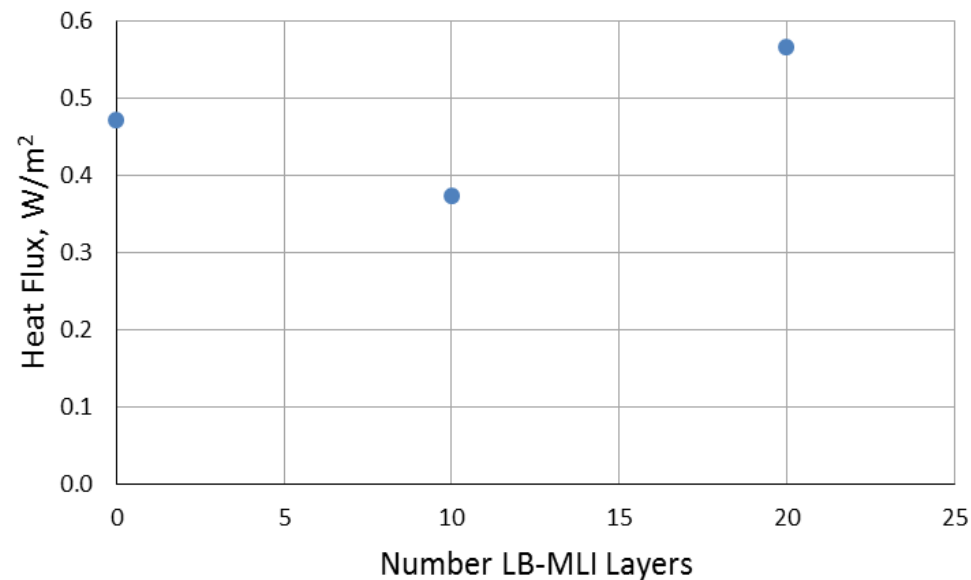


Variable Density MLI:

- Multipurpose Hydrogen Testbed Developed by MSFC
- MLI System designed by Glen McIntosh
 - Foam to prevent liquefaction of air
 - 10 layers at 8 layer/cm
 - 15 layers at 12 layer/cm
 - 16 layers at 16 layer/cm
 - Built using bumper layers to achieve nominal spacing.
- Modify MLI solver to include layer density
 - 10 layers at 7.1 layer/cm (CBT 77 K, WBT 209 K)
 - 15 layers at 11.8 layer/cm (CBT 209 K, WBT 261 K)
 - 20 layers at 14.8 layer/cm (CBT 261 K, WBT 298 K)
- Used Modified Lockheed Equation

Hybrid MLI:

- Developed to take advantage of simpler methods of low density MLI manufacturing
- IMLI and double layer dacron netting
- Testing at KSC



Constant thickness ~ 38 mm (1.5 inches)
Data from A139 (60 layers tMLI) and A142 (20 layers LB-MLI) for 0 and 20 layer LB-MLI



Transient MLI Performance





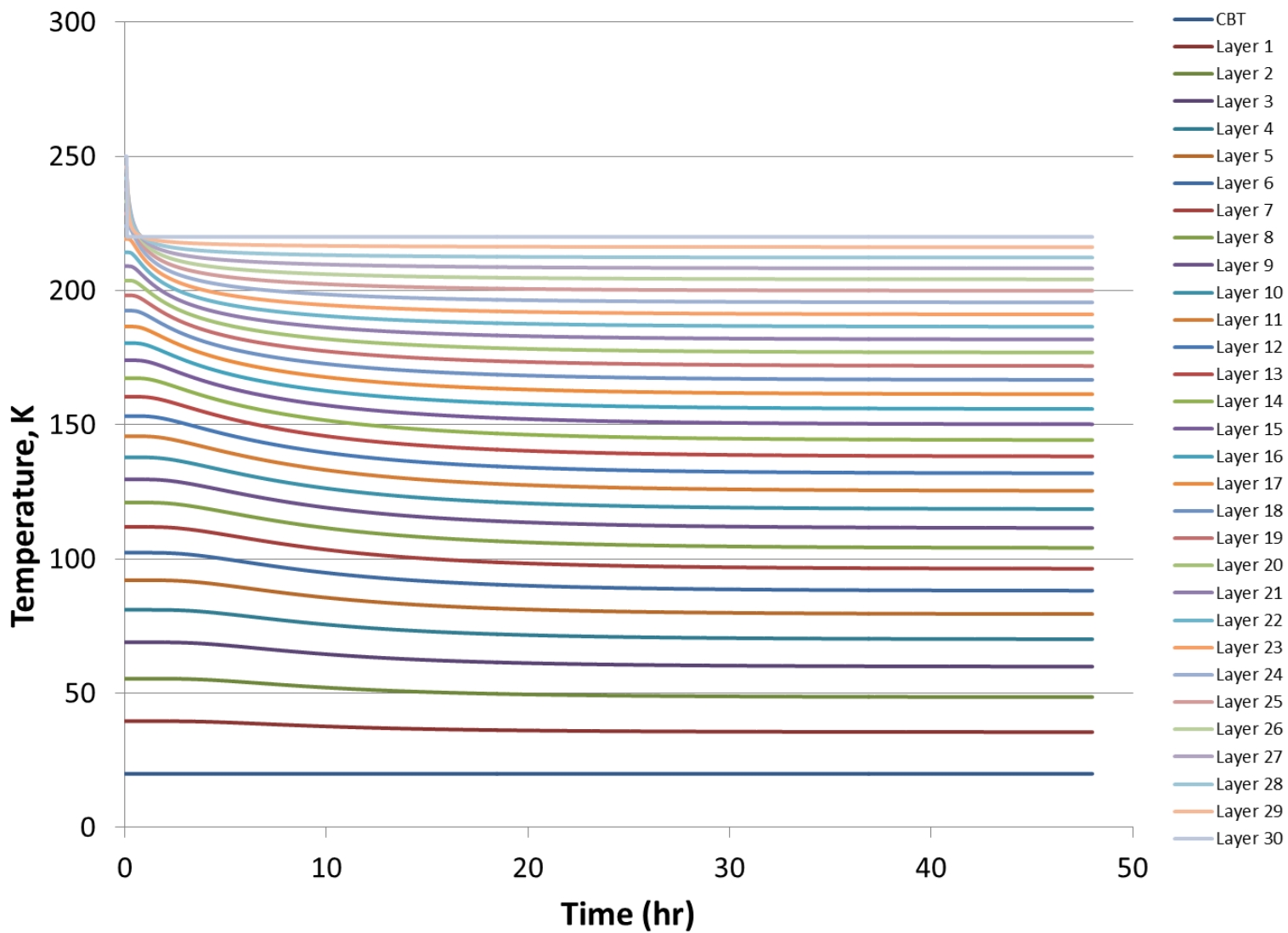
Analysis Method



- **Use Transient MLI code developed under ETDP/CFM**
 - 30 layer blanket at 20 layer/cm
 - Lockheed Report Equation 4-56
 - Calculates temperature of every layer at every time
 - Accounts for thermal mass of each layer & dacron
 - Background pressure of $1 \cdot 10^{-6}$ Torr
- **Start with steady state WBT at T1**
- **After 0.1 hour, drop WBT to T2**
- **Monitor layer temperatures, determine the length of time to reach temp change rate**
- **Two different temperature sets:**
 - Start at 250 K end at 220 K
 - Start at 300 K end at 200 K

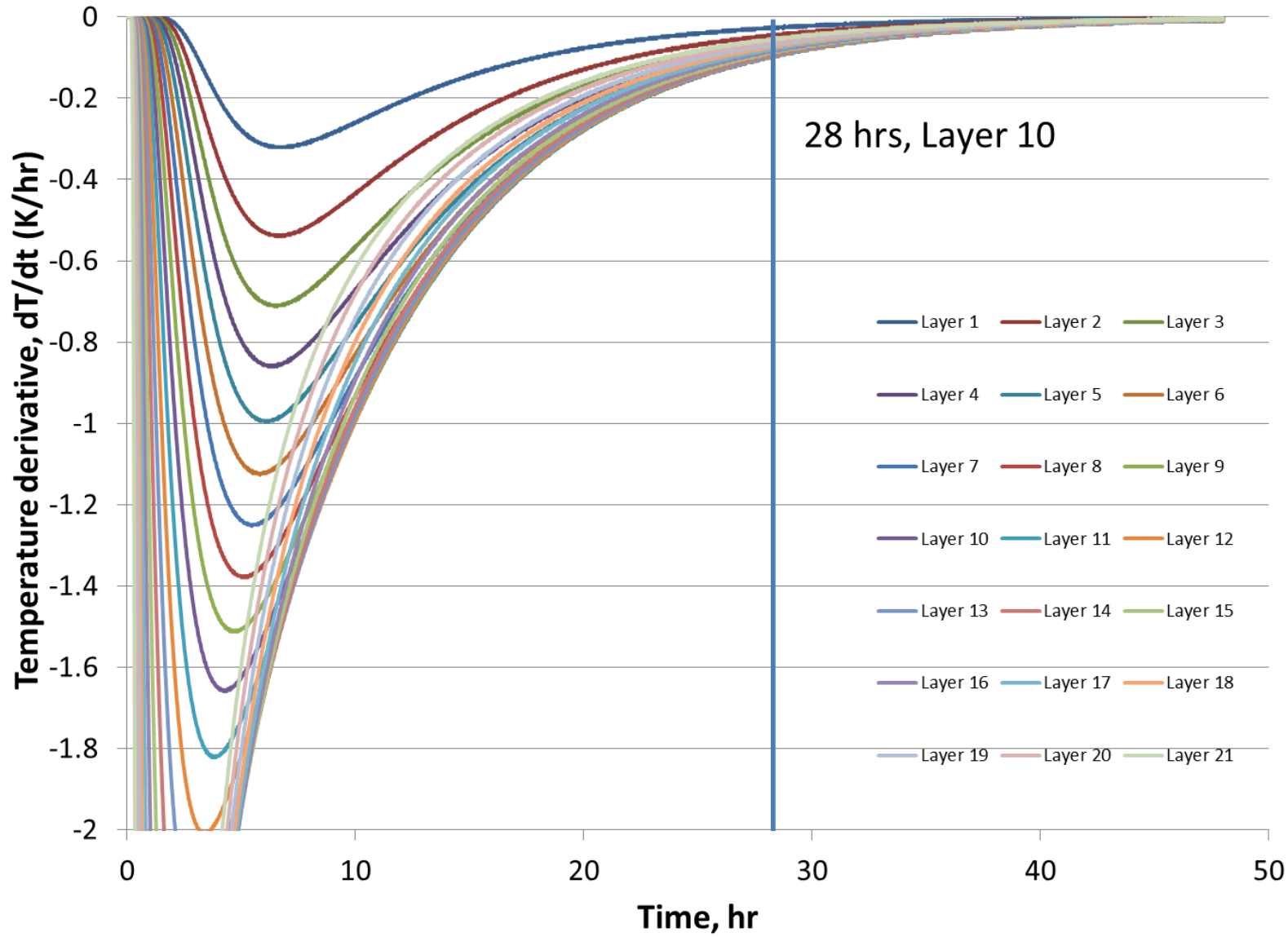


Small Perturbation - Temperatures



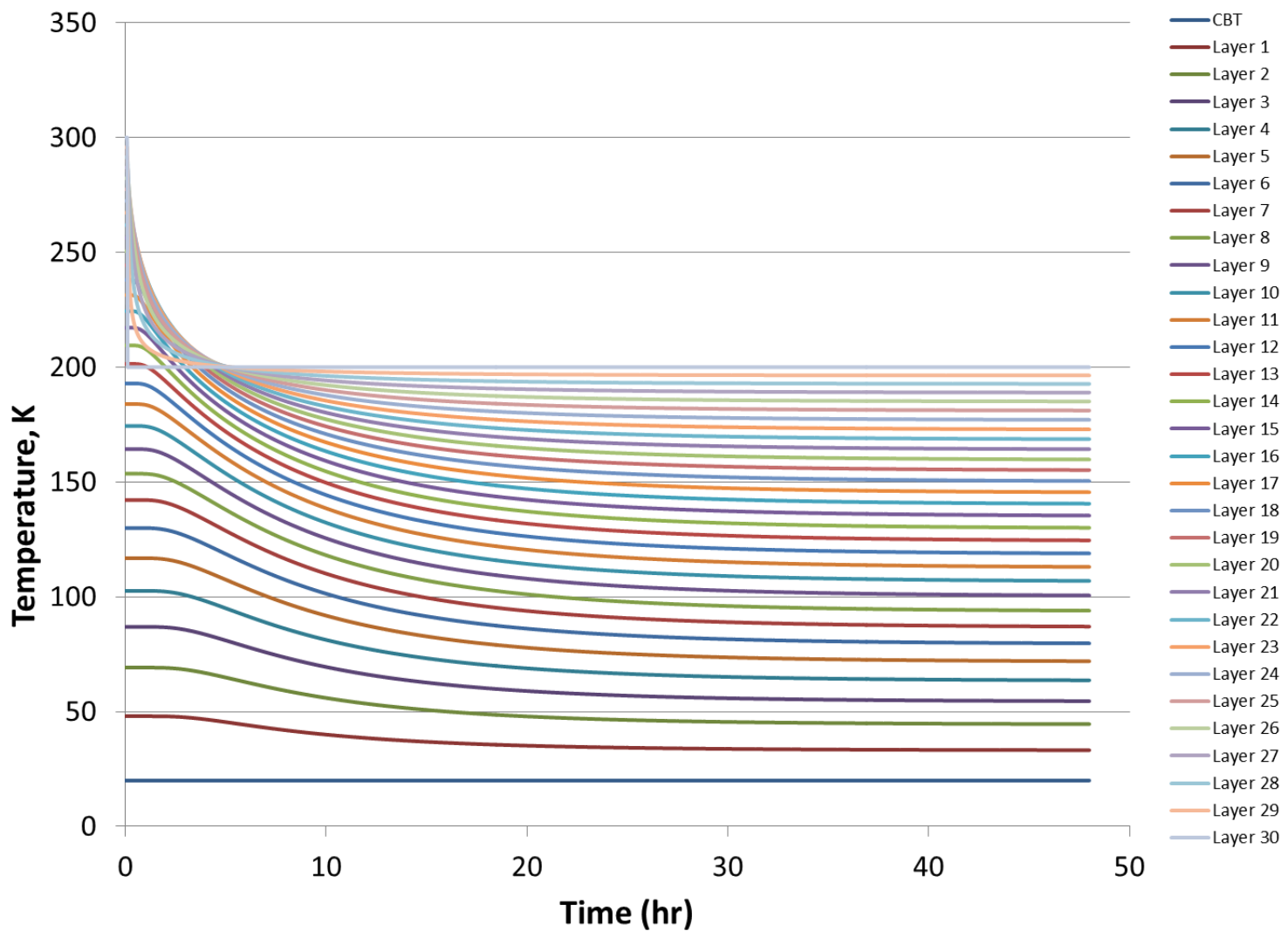


Small Perturbation – Time Derivative



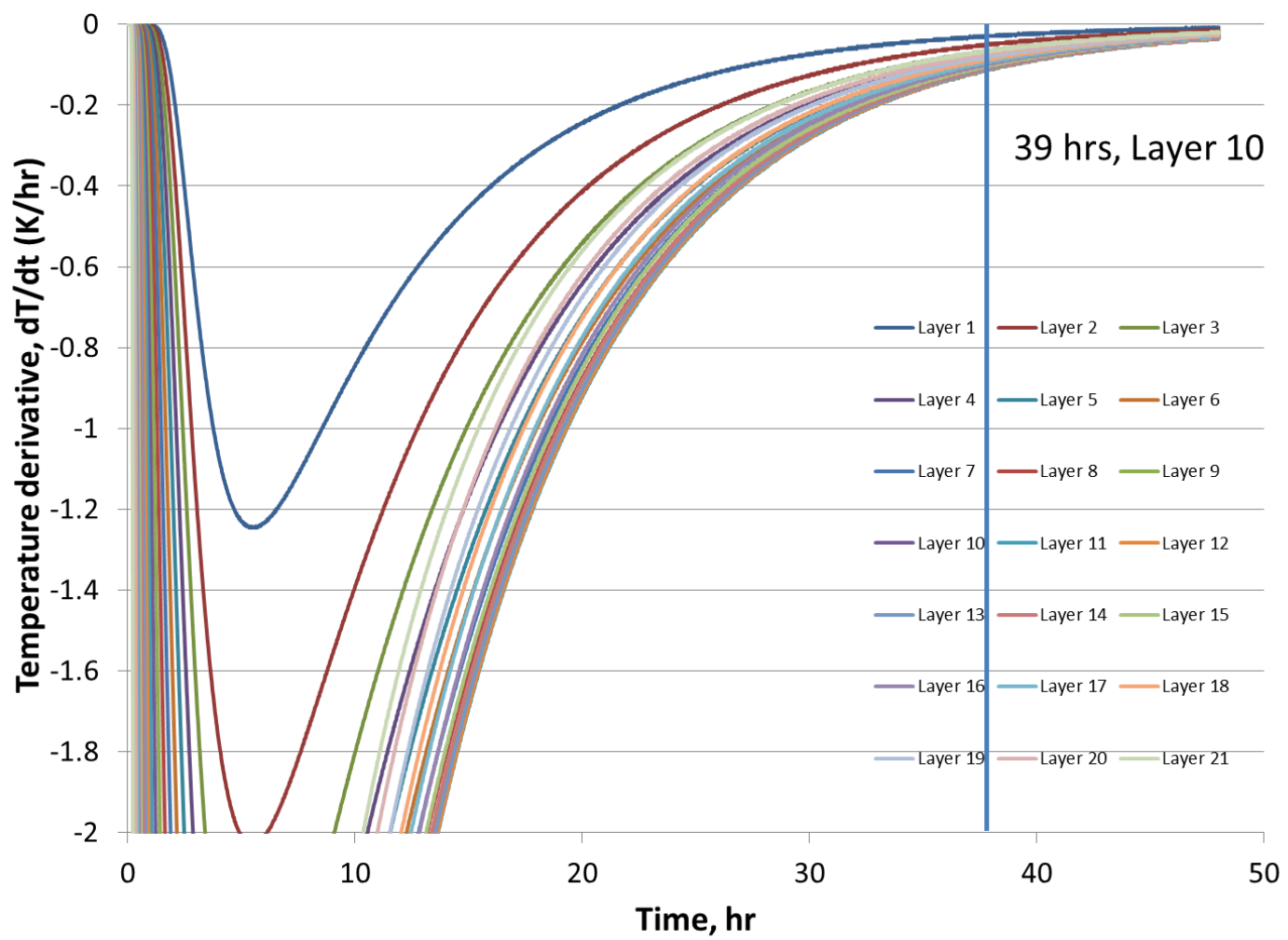


Large Perturbation - Temperatures





Large Perturbation – Time Derivatives





Typical MEO response for LOX tank

Inputs:

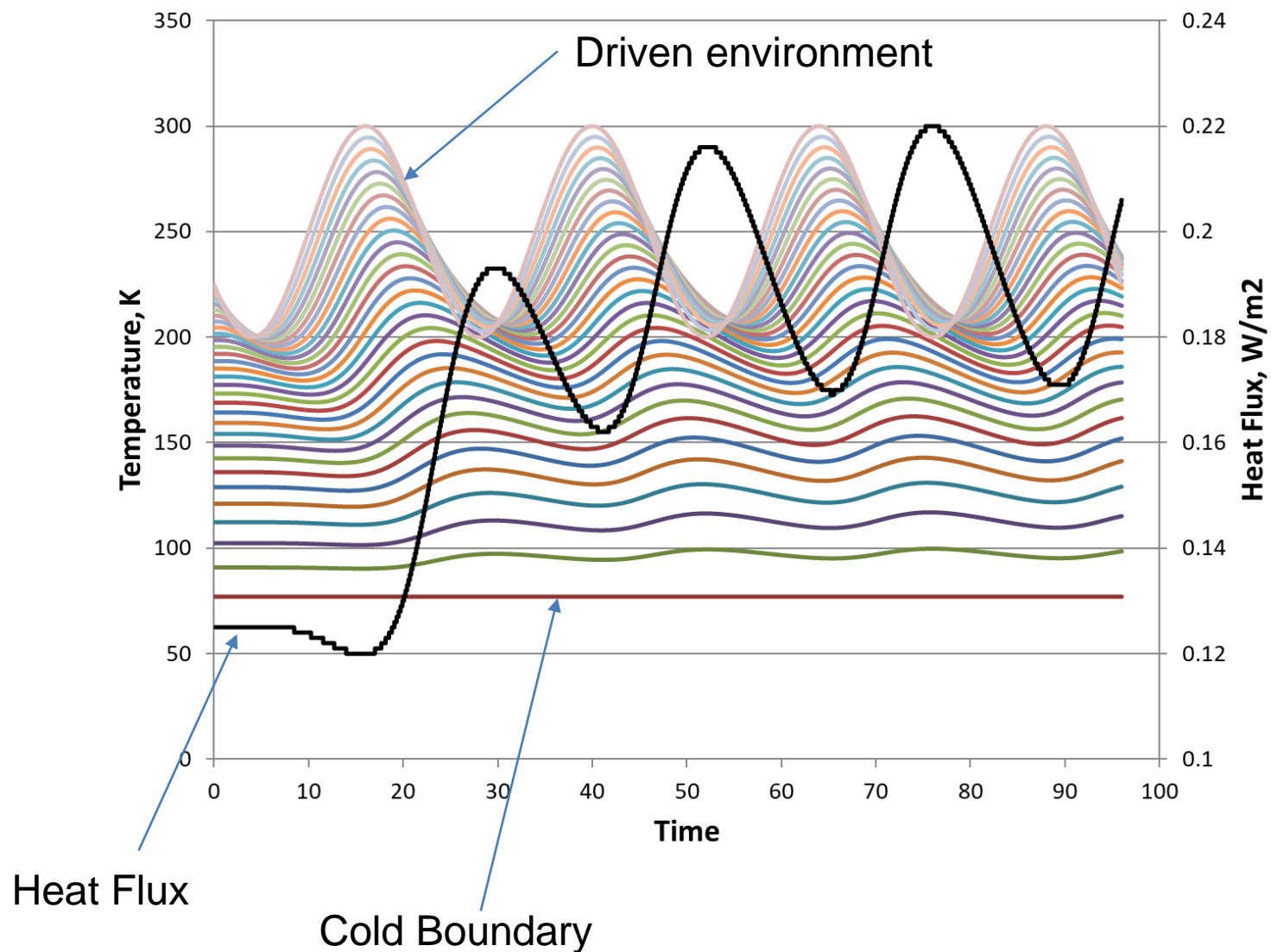
Sinusoidal boundary condition

- Mean Temp ~220 K
- Amplitude ~35 K

30 Reflectors (each on plotted)

3-hour period

Note – I ran all of these cases probably 10 years ago.





Evacuation and transients



First 24 hours

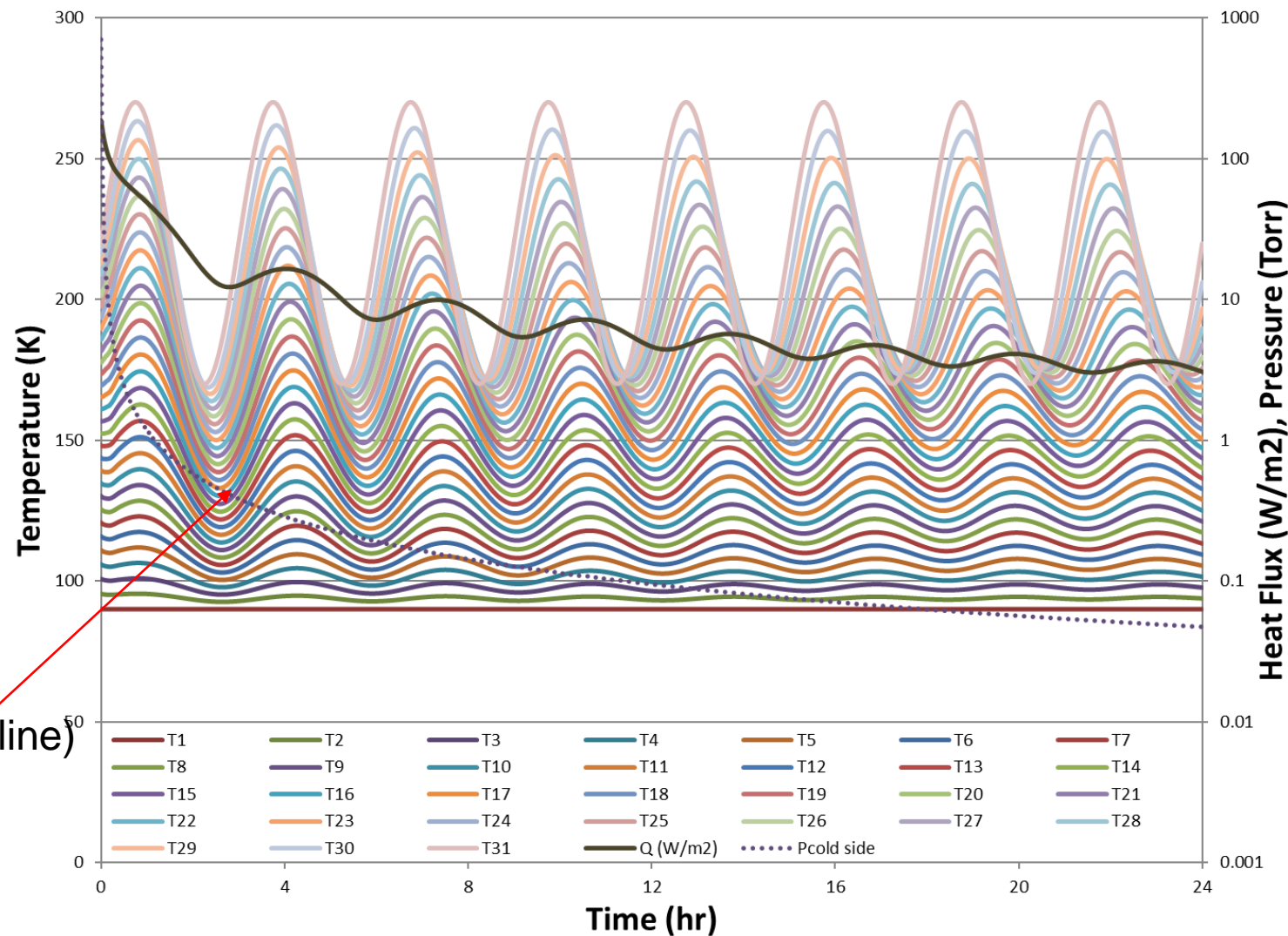
220 K +/- 50 K

Sinusoidal ~ 3 hour period

This run is a combination of evacuation and transient thermal.

I believe the evacuation assumes that all layers are in series. Again, ran it 10 years ago.

Pressure (dotted line)





MLI Materials





Reflectors



- **Aluminum**
 - Generally a foil, much thinner than what you cook with.
 - One side looks shiny, other side looks dull, not indicative of actual IR emissivity.
 - Interaction between aluminum and oxygen creates an oxidation layer on the surface which dramatically impacts the optical properties.
 - Still used in many industrial applications.
 - Not used for spacecraft anymore because the aluminum holds standing waves from various acoustic / vibrational modes within the geometry of the blanket.
- **Gold**
 - A more expensive, but higher performing (lower emissivity) reflector that has been used in lieu of aluminum on a few occasions.
 - There is a significant cause for use at higher temperatures as well.
 - Does not have issues with oxidation.
- **Vacuum Deposited Films**
 - Aluminum, gold, or other highly reflective material deposited via sputtering on to a polymeric film
 - Polyethylene Terephthalate (Mylar), polyimide (Kapton)
 - Can be deposited on a single side or both sides.
 - Typically have radiation transmissive coating to prevent oxidation

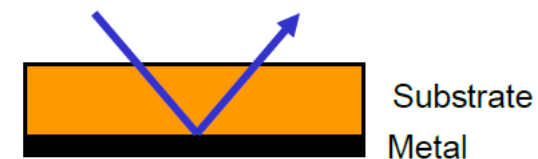
The Folly of Visually Observing Reflectors

- It is very hard to spot 2nd surface reflectors with the naked eye when the substrate is transparent.
- The easiest way to tell is that generally, 1st surface mirrors have a backing on the tape, 2nd surface mirrors don't.
- Substrate is either FEP or Polyimide
- The radiative heat load onto a surface is proportional to the emissivity of the surface.

First Surface Mirrors



Second Surface Mirrors



Metal	Typical Emittance (ϵ)
Gold	0.02
Silver	0.02
Aluminum	0.03

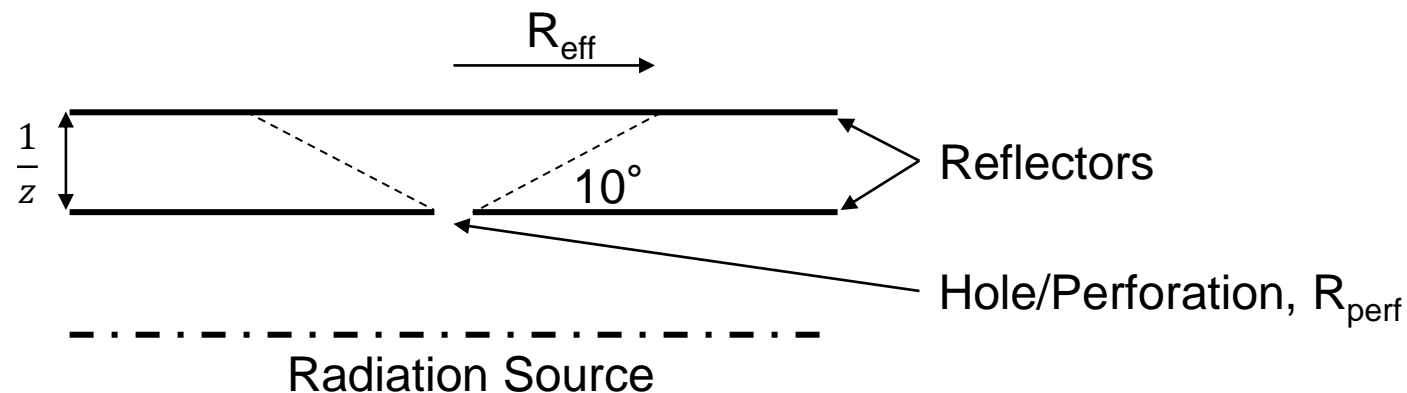
Film Thickness		Typical Emittance	
Mils	Microns	FEP	Polyimide
0.5	12.5	0.41	0.52
1	25	0.52	0.64
2	51	0.65	0.76
5	127	0.79	0.85
10	254	0.86	

$$\dot{Q}_{rad} = \epsilon \sigma A_{surf} (T_H^4 - T_C^4)$$

Images and data from Sheldahl Red Book

Perforations

- **Perforations can be added to a reflector to improve the venting characteristics of a blanket during the launch and ascent sequences.**
- **Perforations allow transmission of radiative energy through the hole with a cone angle of 10 degrees, significantly degrading the blanket performance.**
 - In AIAA paper 73-718, Tien and Cunnington analytically demonstrate that perforation can easily increase the radiative heat transfer portion of the blanket between 30% - 70%.
 - For the same open area percentage, it is better to have larger, more spread-out perforations / holes.
 - Venting analyses can be performed to determine the open area required or if edge venting can be tolerated.



Note in this figure, z is the layer density (using the nomenclature from ASTM C-740)



Spacer Materials

- **No-spacer / dimpling**
 - Dimpling / forming of reflector to minimize contact between layers
 - Other times, for less performing systems, can get away with simple contact resistance between layers
 - Comparative testing has been performed.
- **Tissue Paper**
 - First used material due to low cost and ease of handling.
 - Still used in many industrial applications.
- **Non-woven fiberglass**
 - Fairly cheap, easy to integrate, directly paired with foils or films.
 - Less performance than other spacers, but cheaper than netting and easier to handle.
- **Silk Netting**
 - Used on most early spacecraft due to superior thermal performance than other types of netting
 - High costs and lack of manufacturers in the US have nearly eliminated the use
 - We did do comparative testing at KSC in ~2012.
- **Dacron Netting**
 - Main spacer used on spacecraft for contemporary spacecraft
 - Two forms B2A (less coarse – 2x the mass), B4A (much harder to handle)
- **Other**
 - Superflok – Convair General Dynamics incorporation of specially formed dacron needle shaped spacer tufts
 - xMLI – Quest Thermal Group developed tripod spacers

All spacer types increase the heat load between otherwise not in contact reflective layers

Non-ideal Effects on Multilayer Insulation





MLI Cryogenic System Heat Load Calculator for Trad Cryo MLI



MLI Cryogenic System Heat Load Calculator										Key	Inputs	Selection	Button																																																					
Fluid	Hydrogen																																																																	
Length Dimension	meters																																																																	
Tank Name	1 m tank																																																																	
Tank Diameter	Tank Length	Dome Ratio	Tank Volume	Tank Capacity	Fluid Mass	Tank Surface Area	V:SA	L/D	Tank Capacity Gallons	# Sections	Dome Surface Area	Section SA	Section Length																																																					
1.00	1.36	1.20	0.85	0.81	57.1	4.45	0.19	1.3575	212.9	3	1.40	0.55	0.17																																																					
Areal MLI	NewQ	K	<table border="1"> <thead> <tr> <th colspan="2">Penetrations</th> <th colspan="2">No. of Pins</th> <th colspan="2">Note:</th> </tr> <tr> <th>Shield Emissivity</th> <th>0.03</th> <th></th> <th></th> <th></th> <th></th> </tr> </thead> <tbody> <tr> <td>#</td> <td>Dia (m)</td> <td>dx (m)</td> <td>Heat Load (W)</td> <td colspan="2"></td> </tr> <tr> <td>3</td> <td>0.0254</td> <td>0</td> <td>0.790</td> <td colspan="2">Hanging Points</td> </tr> <tr> <td>1</td> <td>0.11748</td> <td>0</td> <td>0.575</td> <td colspan="2">LH2 Fluid Interface</td> </tr> <tr> <td>1</td> <td>0.11748</td> <td>0</td> <td>0.575</td> <td colspan="2">LN2 Fluid Interface</td> </tr> <tr> <td>0</td> <td>0</td> <td>0</td> <td>0.000</td> <td colspan="2"></td> </tr> <tr> <td colspan="3">Total:</td> <td>1.939</td> <td colspan="2">W</td> </tr> </tbody> </table>										Penetrations		No. of Pins		Note:		Shield Emissivity	0.03					#	Dia (m)	dx (m)	Heat Load (W)			3	0.0254	0	0.790	Hanging Points		1	0.11748	0	0.575	LH2 Fluid Interface		1	0.11748	0	0.575	LN2 Fluid Interface		0	0	0	0.000			Total:			1.939	W		Seams		Butt Seam		Overlap Seam	
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Seams	0.46	W	Pin Radius		3	Structural FEA																																																												
Pins	4.80	W	mm		4	T Environment	259	K																																																										
Other	0	W	mm		4	Number of Nodes	20	K																																																										
Margin	0	%	Layer Spacing		0.33333	Structure Name	#	Structure Material																																																										
Total	12.89	W	Effective Pin Radius		4.9	Coupon	1	Stainless Steel 304																																																										
Total			Pin Spacing		13.8	Aluminum		Aluminum																																																										
Total			Pins per Area		8.10	Aluminum		Aluminum																																																										
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MLI Mass Tool			
Cylindrical with Spherical Heads		Spherical	
1.78	Cylindrical Radius, m	2.25	Spherical Radius, m
1.08	Cylinder Length, m	10	Section 1 MLI layers
0.34	Dome Height, m	0	Section 2 MLI layers
30.00	Section 1 MLI layers	0	Section 3 MLI layers
0.00	Section 2 MLI layers	10	Section 1 MLI lay dens
0.00	Section 3 MLI layers	15	Section 2 MLI lay dens
15.00	Section 1 MLI lay dens	16	Section 3 MLI lay dens
0.00	Section 2 MLI lay dens	0	Foam thickness, m
0.00	Section 3 MLI lay dens		



Penetrations Executive Summary

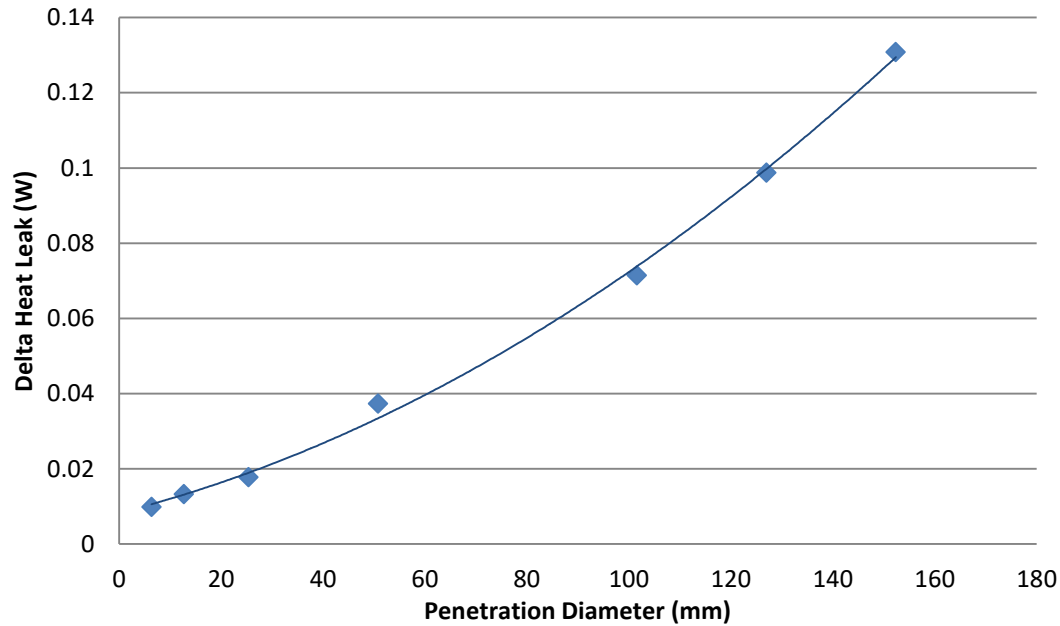
- Testing was performed on a wide variety of penetration integration techniques
- Buffered integration using Cryolite was found to be the best performing integration technique
- A detailed Thermal Desktop-based model was built and validated to the testing data
- Using the same techniques from the detailed thermal model, a more general model was developed to allow for parameterization of the model and understanding how it reacted to the changing of different variables
- Based on the result of the general model, an equation was developed to predict the integration heat load for a given penetration
- Testing was also performed with IMLI, but not included in this analysis. For more information, contact the author.
- Conduction through the penetrations accounted for in addition to this format.

$$dq = q_{ref} \left(\frac{q_{actual}}{q'_{ref}} \right)_{\#layers} \left(\frac{q'_{ref}}{q_{ref}} \right)_{buffer\ thick} \left(\frac{q_{actual}}{q_{ref}} \right)_{diameter} \left(\frac{q_{actual}}{q_{ref}} \right)_{buffer\ thick} \left(\frac{T_h}{297} \right)^{1.56}$$



Model – Scaling - Diameter

Delta Heat Leak vs Penetration Diameter
25 Layers, 6.4mm Cryolite Buffer Thickness



Penetration Details	Change in Heat Leak (W) with Strut Diameter (x in meters)
25 Layers, 6.4mm Cryolite Buffer	$2.95x^2 + 0.346x + 0.00826$

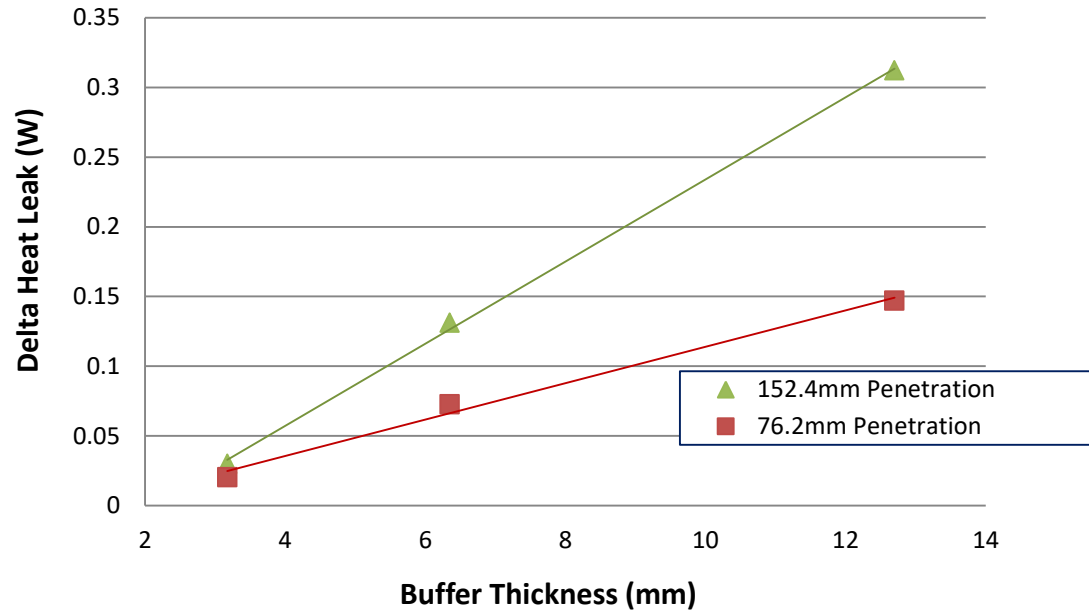
Note: data points shown are from the model and the curve is a curve fit



Model – Scaling – Buffer Thickness

Delta Heat Leak vs Buffer Thickness

25 Layers MLI



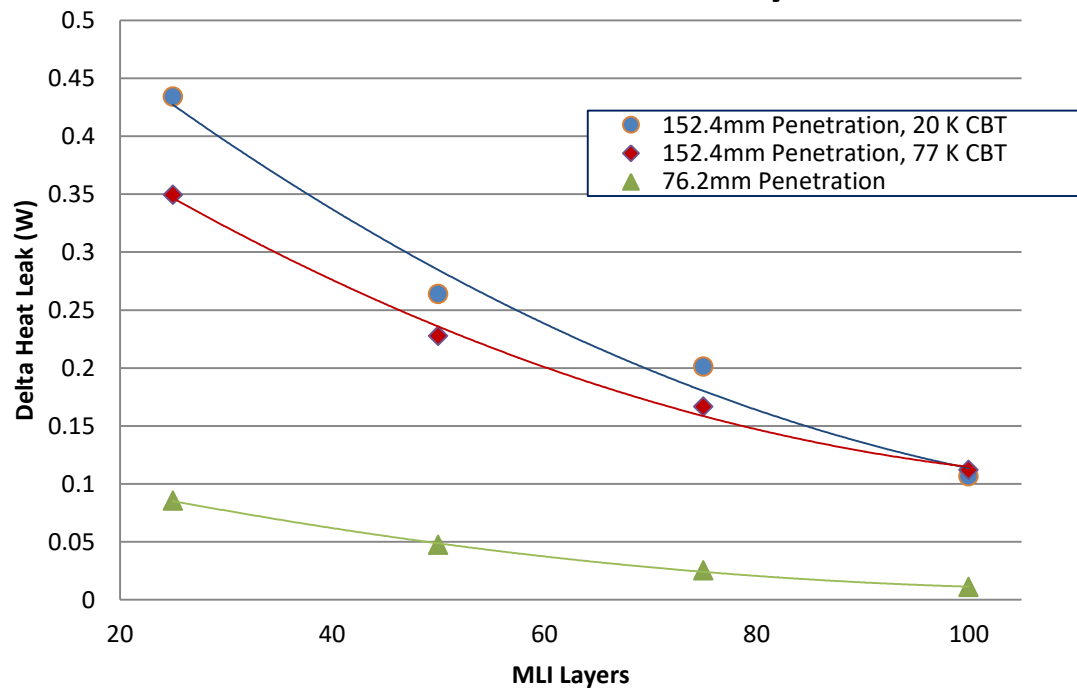
Penetration and Environment	Change in Heat Leak (W) With Buffer Thickness (x in meters)
152.4 mm Strut, 25 Layers MLI	$Y = 29.5x - 0.0608$
76.2 mm Strut, 25 Layers MLI	$Y = 13.1x - 0.0168$



Model – Scaling - # Layers



Delta Heat Leak vs MLI Layers



Penetration Details	Change in Heat Leak (W) With MLI Layers (x)
152.4mm Penetration, 25.4mm Buffer, 20 K Cold Boundary	$3.03E-5x^2 - 7.97E-3x + 0.607$
152.4mm Penetration, 25.4mm Buffer, 77 K Cold Boundary	$2.68E-5x^2 - 6.44E-3x + 0.491$
76.2mm Penetration, 12.7mm Buffer	$9.51E-6x^2 - 2.17E-3x + 0.134$



Results – Model Summary

- As a result of the model scaling, a multipart equation was developed
- Considers warm boundary temperature, MLI system # of layers, penetration diameter, and buffer thickness

$$dq = q_{ref} \left(\frac{q_{actual}}{q'_{ref}} \right)_{\#layers} \left(\frac{q'_{ref}}{q_{ref}} \right)_{buffer\ thick} \left(\frac{q_{actual}}{q_{ref}} \right)_{diameter} \left(\frac{q_{actual}}{q_{ref}} \right)_{buffer\ thick} \left(\frac{T_h}{297} \right)^{1.56}$$

- Requires the use of two reference states
 - Recommend:
 - Reference 1: 0.0762 m diameter penetration with 25 layers MLI, and 0.0064 m buffer
 - Reference 2 (or prime): 0.0762 m diameter, 25 layers, 0.0127 m buffer
 - Alternate: use 0.1524 m diameter penetration for both with same other variables



Results – Model Sample Calculation

Calculate the degradation due to a 104 mm (4 inch) pipe going through 60 layers of MLI using an 8 mm (~0.75 inch) Cryolite buffer with a warm boundary temperature of 297 K.

$$dq = q_{ref} \left(\frac{q_{actual}}{q'_{ref}} \right)_{\#layers} \left(\frac{q'_{ref}}{q_{ref}} \right)_{buffer\ thick} \left(\frac{q_{actual}}{q_{ref}} \right)_{diameter} \left(\frac{q_{actual}}{q_{ref}} \right)_{buffer\ thick} \left(\frac{T_h}{297} \right)^{1.56}$$

- For reference case one use 25 layers of MLI with a 76.2 mm penetration and a 6.4 mm Cryolite buffer. Qref equals 0.052 W from Slide 46.
- For reference case two use 25 layers of MLI with a 76.2 mm penetration and a 12.7 mm Cryolite buffer. Qref' then equals 0.086 W from Slide 48.
- Q actual for the pipe diameter (using a 104 mm penetration with 25 layers of MLI & 6.4 mm Cryolite buffer) is 0.076 W from Slide 46.
- Q actual for the buffer thickness (using an 8 mm Cryolite buffer with 76.2 mm penetration and 25 layers) is 0.088 W from Slide 47.
- Q actual for the number of layers (using 60 layers with a 12.7 mm buffer and a 76.2 mm penetration) is 0.038 W from Slide 48.
- Since the WBT is 297 K, we can neglect the last term as 1

$$\begin{aligned} dq &= 0.052 \left(\frac{0.038}{0.086} \right)_{\#layers} \left(\frac{0.086}{0.052} \right)_{buffer\ thickness} \left(\frac{0.076}{0.052} \right)_{diameter} \left(\frac{0.088}{0.052} \right)_{buffer\ thickness} \\ &= 0.095W \end{aligned}$$



MLI Cryogenic System Heat Load Calculator for Trad Cryo MLI



MLI Cryogenic System Heat Load Calculator										Key	Inputs	Selection	Button
Fluid	Length Dimension	Hydrogen											
	Tank Name	meters											
		1 m tank											
Tank Diameter	Tank Length	Dome Ratio	Tank Volume	Tank Capacity	Fluid Mass	Tank Surface Area	V:SA	L/D	Tank Capacity Gallons	# Sections	Dome Surface Area	Section SA	Section Length
1.00	1.36	1.20	0.85	0.81	57.1	4.45	0.19	1.3575	212.9	3	1.40	0.55	0.17
Areal MLI	NewQ	K	Penetrations										
WBT	300	K	No Buffer										
CBT	20	K	Shield Emissivity										
# Layers	60	lay/cm	#	Dia (m)	dx (m)	Heat Load (W)	Note:						
Layer Density	30	lay/cm	3	0.0254	0	0.790	Hanging Points						
Vac Press	5E-06	Torr	1	0.11748	0	0.575	LH2 Fluid Interface						
C _a Coefficient	1.05E-05		1	0.11748	0	0.575	LN2 Fluid Interface						
Areal Heat Load	1.28	W/m ²	0	0	0	0.000	Total: 1.939 W						
SF	5.69	W	Penetrations Calc										
	1		Areal MLI Calc										
Total			Pins										
Blankets	5.69	W	degree	10									
Penetrations	1.94	W	radians	0.175									
Seams	0.46	W	Pin Radius	3	mm	Hole Radius	4	mm					
Pins	4.80	W	Layer Spacing	0.33333	mm	MLI Thickness	2.00	cm					
Other		W	Effective Pin Radius	4.9	mm	Pin Material	IM7-8552						
Margin	0	%	Pin Spacing	13.8	in	Pins per Area	8.10	Number Pins/m ²					
Total	12.89	W	Effective Flux	1.08	W/m ²	Total Heat	4.80	W	Pin Calc				
Seams													
Butt Seam						Overlap Seam							
n-Layers	60		Length	10.82	m								
Layer-Rho	30	layer/cm	Layer Density	60	lay/cm								
Length	10.55	m	Overlap Width	2	in								
Width	0.002	m	Qcalc	3.678	W/m ²								
Depth	0.020	m	Seam Area	0.550	m ²								
f	100.9		Seam Load	2.023	W								
Stefan-Boltzman	5.67E-08	W/m ² -K ⁴	Constant										
Th	300	K	Degradation	8%									
Tc	20	K	Heat Flux	5.69									
qdot	0.235	W/m	Seam Load	0.46	W								
Seam Load	2.481	W	Hinckley, R.B., <i>Liquid Propellant Losses During Space Flight, Final Report</i> . NASA-CR-53336, Arthur D. Little, Inc. Cambridge, MA, 1964.										
Structural FEA													
T Environment	259	K	WBT	300	K	FEA Calc							
Number of Nodes	20		CBT	20	K								
Structure Name	#	Structure Material	Emissivity	Structure Length (m)	Structure Area (m ²)	Q _{total} (W)							
Coupon	1	Stainless Steel 304	0.3	0.750	0.002	8.25							
		Aluminum				0.00							
		Aluminum				0.00							
		Aluminum				0.00							
MLI Mass Tool													
Cylindrical with Spherical Heads						Spherical							
1.78	Cylindrical Radius, m	2.25	Spherical Radius, m										
1.08	Cylinder Length, m	10	Section 1 MLI layers										
0.34	Dome Height, m	0	Section 2 MLI layers										
30.00	Section 1 MLI layers	0	Section 3 MLI layers										
0.00	Section 2 MLI layers	10	Section 1 MLI lay dens										
0.00	Section 3 MLI layers	15	Section 2 MLI lay dens										
15.00	Section 1 MLI lay dens	16	Section 3 MLI lay dens										
0.00	Section 2 MLI lay dens	0	Foam thickness, m										
0.00	Section 3 MLI lay dens												
OxINS	Helms	MethaneINS	HydrogenINS	Pins	Penetrations	Tank Size	Other heat Loads (He)	Other Heat ...					



Attaching MLI Blankets: Is this really an issue?

ATV 1 (Joules Verne) incident

- During launch, more power draw required than expected, was traced to blanket disengagement.
- Root causes came down to improper structural attachment
- AIAA-2010-6197





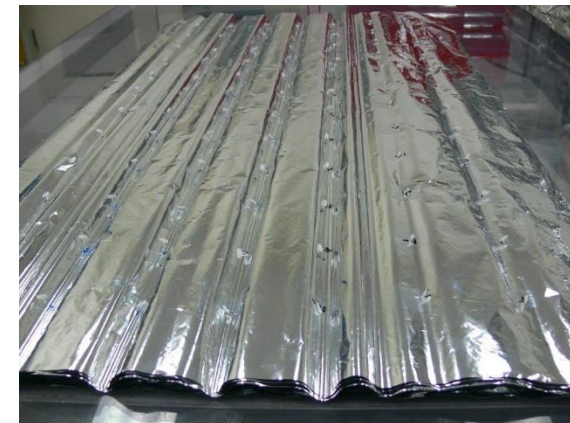
Nylon Tag Testing

- **Nylon tags have long been used to hold MLI together**
- **Installed 56 pins into an existing 10 layer LB-MLI blanket**
 - Individual pins have a really small heat load (~0.9 mW each)
 - Needed repeatable MLI coupon to do initial test and pinned test
 - Pin spacing ~ 3 inch
- **Blanket Heat flux (KSC – Cryostat 100):**
 - A164 July 2012¹: 0.92 W/m²
 - A191 March 2015: 1.04 W/m²
 - Was also used in Hybrid MLI testing² (A174, A175, A181, A182)
- **Predicted disturbance:**
 - Variable tag geometry
 - 20 node conduction model (NIST nylon props): 0.5 mW/tag
 - Direct radiation through hole: 8 μW/tag

Cold side



Hot side



¹Johnson, W.L., Heckle, K.W., and Hurd, J. “**Thermal coupon testing of Load-Bearing Multilayer Insulation**”, *AIP Conference Proceedings* 1573, pg. 725, 2014.

²Johnson, W.L., Fesmire, J.E., and Heckle, K.W., **Demonstration of Hybrid Multilayer Insulation of Fixed Thickness Applications**, *IOP Conf. Ser.: Mater. Sci. Eng.* 101 012015, 2015.



Nylon Tag Test Results Analysis



- **Total heat to the blanket (with 56 tags): 0.51 W**
 - 0.35 W through blanket
 - 0.16 W (+/- 0.025) residual (i.e. through tags)
- **Predicted load: 45 mW**
- **Measured heat load is 3.5 x predicted heat load**

- **Similar to Arthur D. Little, Inc results from 1966³**
 - Single 0.8 mm nylon pin through 10 layers MLI (1.0 mm diameter hole)
 - Predicted heat load of 0.3 mW
 - Measured change in heat load of ~ 3 mW, which was the experimental error

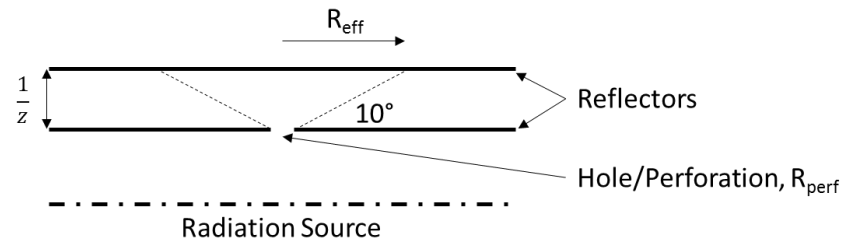
- **Need revised model**

³Black, I.A, Glaser, P.E., Reid, R.C., “Heat Loss Through Evacuated Multilayer Insulation Penetrated By a Low-Conductivity Pin”, Bull. IIR, Annex 1966-2, 233-243 (Meeting Of Commission 2, Trondheim, Norway, Jun 22-24, 1966)



Small Penetration Revised model

- Based on perforations model developed for MHTB large perforations, the radiation through a perforation is not limited to direct radiation⁴
- Instead the effective radiation area is defined by a 10 deg angle
- Using layer density as the spacing for LB-MLI, this can be extrapolated to a tag hole.



$$\theta = 10 \text{ deg} = 0.175 \text{ rad}$$

$$r_{eff} = \frac{1}{z \cos \theta} + r_{perf}$$

$$A_{eff} = \pi r_{eff}^2$$

$$\dot{Q} = A_{eff} \epsilon_{layer} \sigma (T_h^4 - T_c^4) + \int \frac{A}{dx} \int k dT$$

- Revised model estimates 3.6 mW per tag on recent testing (~30% more than actual)
- Revised model estimates 3.6 mW heat load for tag & hole in ADL test

⁴Fox, E.C., Keifel, E.R., and McIntosh, G.L., et.al. "Multipurpose Hydrogen Test Bed System Definition and Insulated Tank Development", Martin Marietta Astronautics, NASA CR-194355, July 1993.



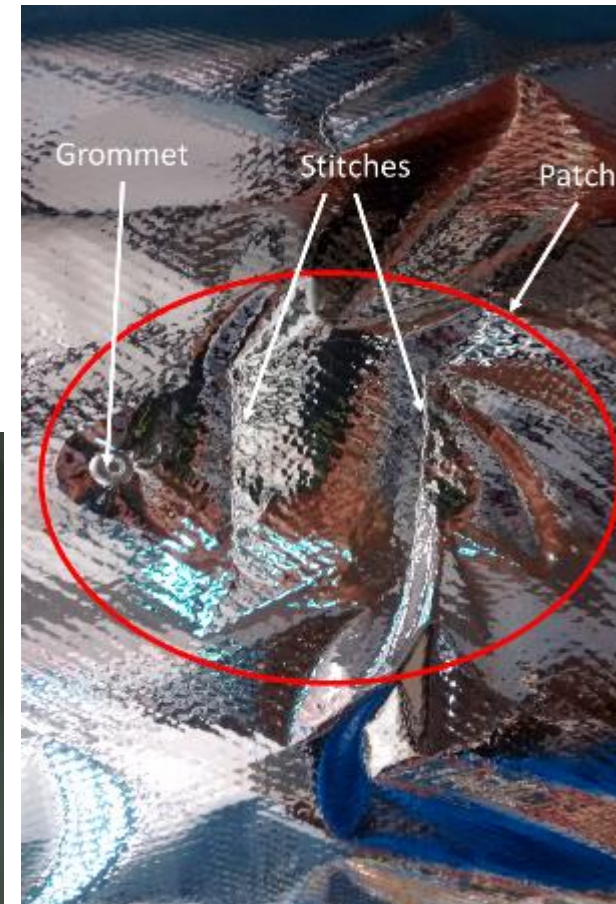
SHIVER Structural Attachments



Test article included 4 patches as shown in the pictures to the right.

Stitching only in outer sub-blanket

Thermocouples on the inside of the blanket as shown below.





SHIVER Attachment Thermal Results



Thermal Testing:

Configuration	Q_{total} , W	Q_{net} , W	Q_{attach} , W	WBT, K	T_{stitch} , K	$T_{blanket}$, K
Baseline	0.928	0.923		261		
Attachments	0.979	0.974	0.052		235	207

Thermal Penalty of 52 mW for four patches

Thermal Modeling:

Heat Loads for one stitch or hole	WBT = 260 K CBT = 230 K	WBT = 260 K CBT = 200 K	WBT = 260 K CBT = mixed*
Conduction (mW/thread)	0.10	0.21	0.10
Radiation (mW) all holes	3.4	5.7	5.7
Total (mW)	11.8	22.6	13.8

*Radiation cold boundary of 200 K, conduction cold boundary of 230 K

Thermal Penalty of 55 mW for four patches



Small Penetration Summary

- **Completed testing on an MLI blanket with multiple small penetrations.**
- **Results show that heat load much more than conduction only.**
- **Analytical approach with combined radiation and conduction shows uncertainty less than 30%.**
 - Change in vacuum level may account for difference
- **Verified model approach for SHIVER test articles in predicting SHIVER heat loads.**

Test Series	Hole Radius (mm)	# layers	Layer Density (lay/mm)	Q_{hole} (mW)	Q_{pin} (mW)	Q_{total} (mW)	Q_{meas} (mW)
A192	0.5	10	0.6	3.1	0.52	3.6	2.0-2.8
Black [9]	0.5	10	1.3	3.3	0.3	3.6	~3

From: Johnson, W.L., Heckle, K.W., and Fesmire, J.E., Heat Loads Due to Small Penetrations in Multilayer Insulation Blankets, IOP Conf. Series: Materials Science and Engineering 278 012197, 2017.

Johnson, W.L., Oberg, D., Frank, D., Mistry, V, and Koci, F.D., Testing of SHIVER MLI Coupons for Heat Load Predictions, IOP Conf. Series: Materials Science and Engineering 755 012151, 2020.

9. I.A. Black, P.E. Glaser, and R.C. Reid, “Heat Loss Through Evacuated Multilayer Insulation Penetrated By a Low-Conductivity Pin”, Bull. IIR, Annex 1966-2, 233-243 (Meeting Of Commission 2, Trondheim, Norway, Jun 22-24, 1966).



MLI ATTACHMENTS

Small Penetration, MLI Standoff, Structural Testing

- In order to attach MLI blankets to spacecraft in a manner to survive a combination of acceleration, acoustic, and venting loads while minimize the parasitic heat load to the tank.
 - The attachments serve as direct heat loads to the tank and often are a significant portion of a tank applied heat load [1].
 - There have been instances where MLI was not appropriately attached to spacecraft and has been lost or damaged, compromising the mission. [2, 3].
 - Based on a review of typical attachment methods, most use plastic (nylon or ultem) holders to minimize conduction loss through a blanket. However, these plastics have a much larger coefficient of thermal contraction and often contract 1% or more than most base metals [4]. As such, an epoxy must be able to handle the differential contraction between the two materials and also handle the many other forces that it may encounter.
- A typical insulation system for a cryogenic upper stage would include spray-of foam insulation (SOFI) underneath the MLI blankets to prevent air liquefaction.
 - The polyetherimide standoffs would be attached to the tank and protrude through the SOFI to provide points of attachment for the MLI blankets. Previous attempts to attach the MLI directly to SOFI has induced cracking in the SOFI as shown in Figure 1.
- For reference, Figure 2 shows a possible configuration of a standoff with an MLI blanket.

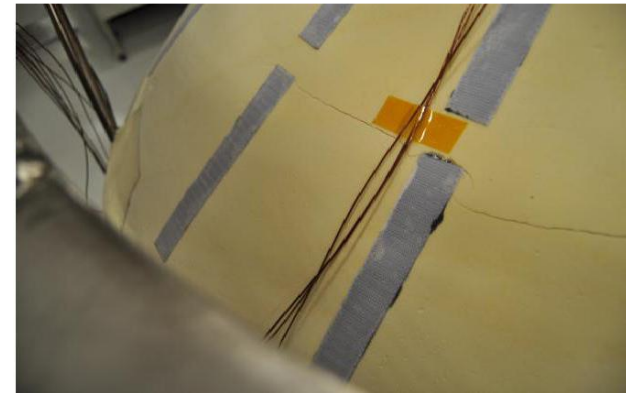


Figure 1. SOFI after MLI was directly attached to the surface.

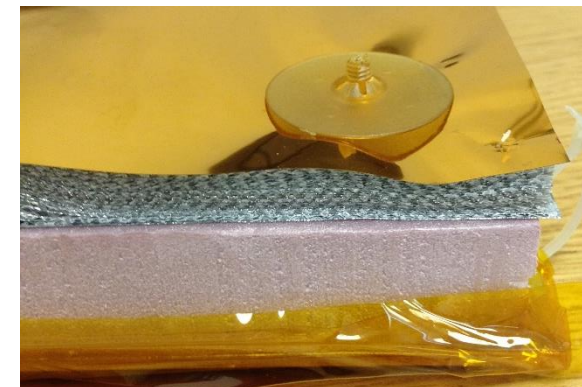


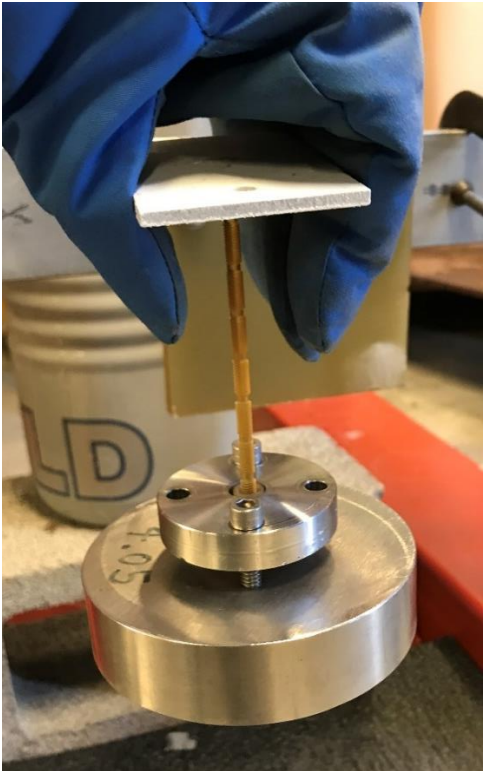
Figure 2. An MLI standoff holding the MLI blanket and foam insulation to the metal surface.



Coupon Tensile and Shear Testing Method

Shear Tests

Pull Tests





Epoxy Tensile and Shear Testing Results



✓ Epoxy success
✗ Epoxy failure (EF)
- Standoff Break (SB)
* Popping noises while cooling
** Sound observations not recorded

Epoxy Brand	Coupon Material	Trial	Popping	Cryogenic Tensile Epoxy Testing										Cryogenic Shear Epoxy Testing									
				Weight, lbs (Coupons maintained at approx. -187°C)																			
				2.9	4.5	8.0	11.6	15.2	17.6	23.1	26.1	35.4	40.8	2.5	4.1	7.6	11.2	14.8	17.2	22.7	25.7	35.0	40.4
Altheris	C1	AL - 2195	1	*	✓	✓	✓	✓	✓	✓	✓	✓	✗ Epoxy failure										
		AL - 2195	2	*	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✗ EF	
	SST 304	1	*	✓	✓	✗ Epoxy failure																	
		2	*	✗ Epoxy failure																			
	EA-2A	AL - 2195	1		✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓
		AL - 2195	2		✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓
SST 304	1		✗ Epoxy failure																				
	2		✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	
CTD	621	AL - 2195	1		✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	
		primered	2		✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	
		AL - 2195	1		✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	
		SST 304	1		✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	
		2		✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓		
		2		- Standoff break																			
GE	Varnish	AL - 2195	1		✗ Epoxy failure																		
		AL - 2195	1	**	✗ Epoxy failure																		
		SST 304	1		✗ Epoxy failure																		
Huntsman	EPIBond	AL - 2195	1	*	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	
		primered	2	*	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	
		AL - 2195	1	*	✗ Epoxy failure																		
		SST 304	1	*	✗ Epoxy failure																		
		2	*	✓	✗ Epoxy failure																		
HYSOL	EA 9430	AL - 2195	1		✗ Epoxy failure																		
		AL - 2195	1	**	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	
		SST 304	1		✗ Epoxy failure																		
	EA 9369	AL - 2195	1		✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	
		primered	2		✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	
		AL - 2195	1	*	✗ Epoxy failure																		
		1	*	✓	✗ Epoxy failure																		
SST 304	1		- Standoff break																				
2		✗ Epoxy failure																					

Additional Surface Preparation

Great Performance



Epoxy Tensile and Shear Testing Results (cont.)

✓ Epoxy success X Epoxy failure (EF) - Standoff Break (SB) * Popping noises while cooling ** Sound observations not recorded

Epoxy Brand	Coupon Material	Trial	Popping	Cryogenic Tensile Epoxy Testing													Cryogenic Shear Epoxy Testing							
				Weight, lbs (Coupons maintained at approx. -187°C)																				
				2.9	4.5	8.0	11.6	15.2	17.6	23.1	26.1	35.4	40.8	2.5	4.1	7.6	11.2	14.8	17.2	22.7	25.7	35.0	40.4	
Masterbond	AE-10	AL - 2195	1	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓			
		2	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓			
		SST 304	1	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	- SB		
	2	* X Epoxy failure																						
	EP29LPSP	AL - 2195 primered	1	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓		
		2	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	- SB		
		3	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓		
		AL - 2195	1	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓		
		1	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓		
		SST 304	2	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓		
	3	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓			
	EP2TIGHT-1	AL - 2195 primered	1	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓		
		2	✓	✓	✓	✓	- Standoff break																	
		SST 304	1	** X Epoxy failure																				
		2	** X Epoxy failure																					
	EP30-2	AL - 2195 primered	1	✓	✓	✓	✓	X Epoxy failure																
		2	✓	✓	✓	X Epoxy failure																		
		SST 304	1	X Epoxy failure																				
		2	X Epoxy failure																					
	SUP12AQHT-LO	AL - 2195 primered	1	✓	✓	✓	X Epoxy failure																	
2		✓	X Epoxy failure																					
SST 304		1	X Epoxy failure																					
2		X Epoxy failure																						
Scotchweld	2216	AL - 2195 primered	1	**	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓			
		AL - 2195	1	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓			
		SST 304	1	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	Missing coupon		
STYCAST	2850FT	AL - 2195 primered	1	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓			
		2	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓			
		AL - 2195	1	* ✓	✓	X Epoxy failure																		
		SST 304	1	* ✓	✓	X Epoxy failure																		
	2	✓	✓	✓	X Epoxy failure																			
	1266	AL - 2195 primered	1	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓		
		2	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓		
		AL - 2195	1	X Epoxy failure																				
SST 304		1	* ✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	X Epoxy failure			
2	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	✓	X Epoxy failure				

Additional Surface Preparation

Great Performance

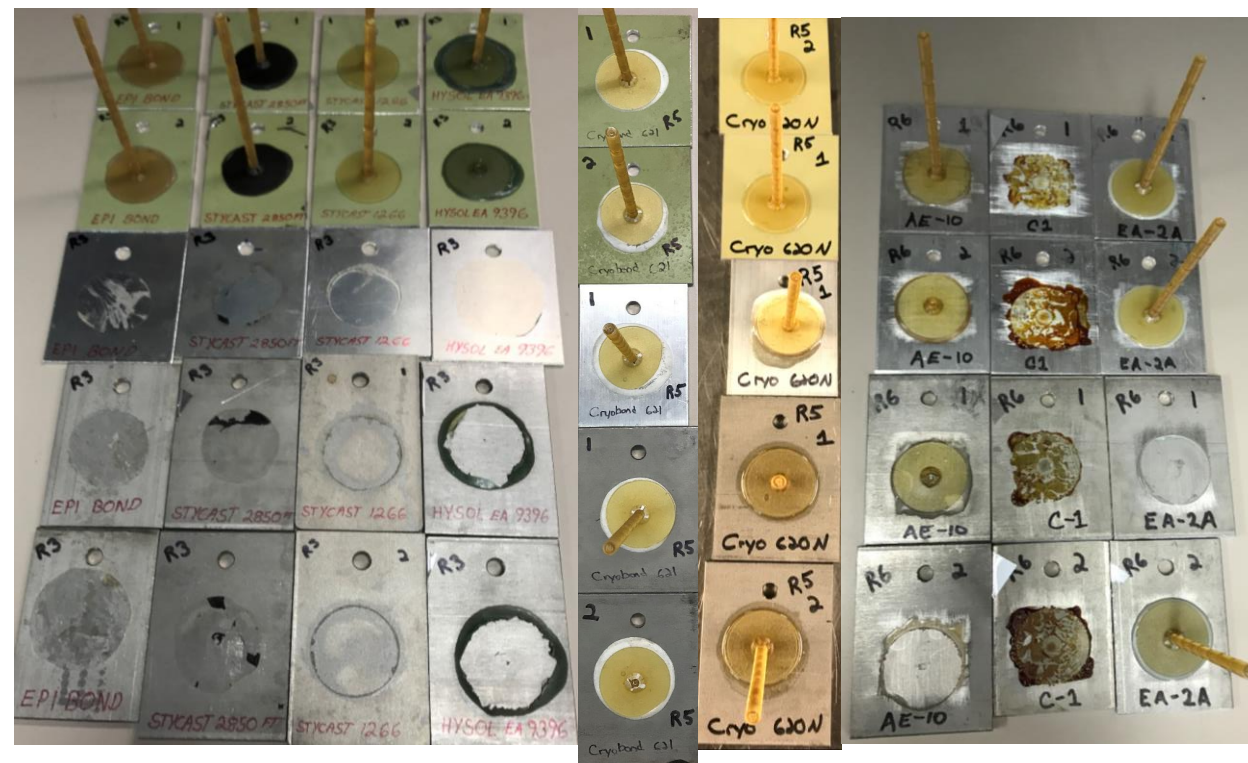
Great Performance



Conclusions/Results

- Best performance from **CTD CryoBond 621**, **Masterbond EP29LPSP**, and **Scotchweld 2216**
 - Passed tensile and shear testing for multiple coupon materials
- Sound indicators of epoxy and standoff failures
- 5 of 6 standoff failures occurred above 25lbs
- 24 of 65 samples survived tensile and shear testing (not including standoff breaks)

Round 3, 5, & 6 coupons after testing



Alberts, S.J., Doehne, C.J., and Johnson W.L., **Testing Tensile and Shear Epoxy Strength at Cryogenic Temperatures**, presented at the 2017 Cryogenic Engineering Conference, Madison, WI, June 2017.

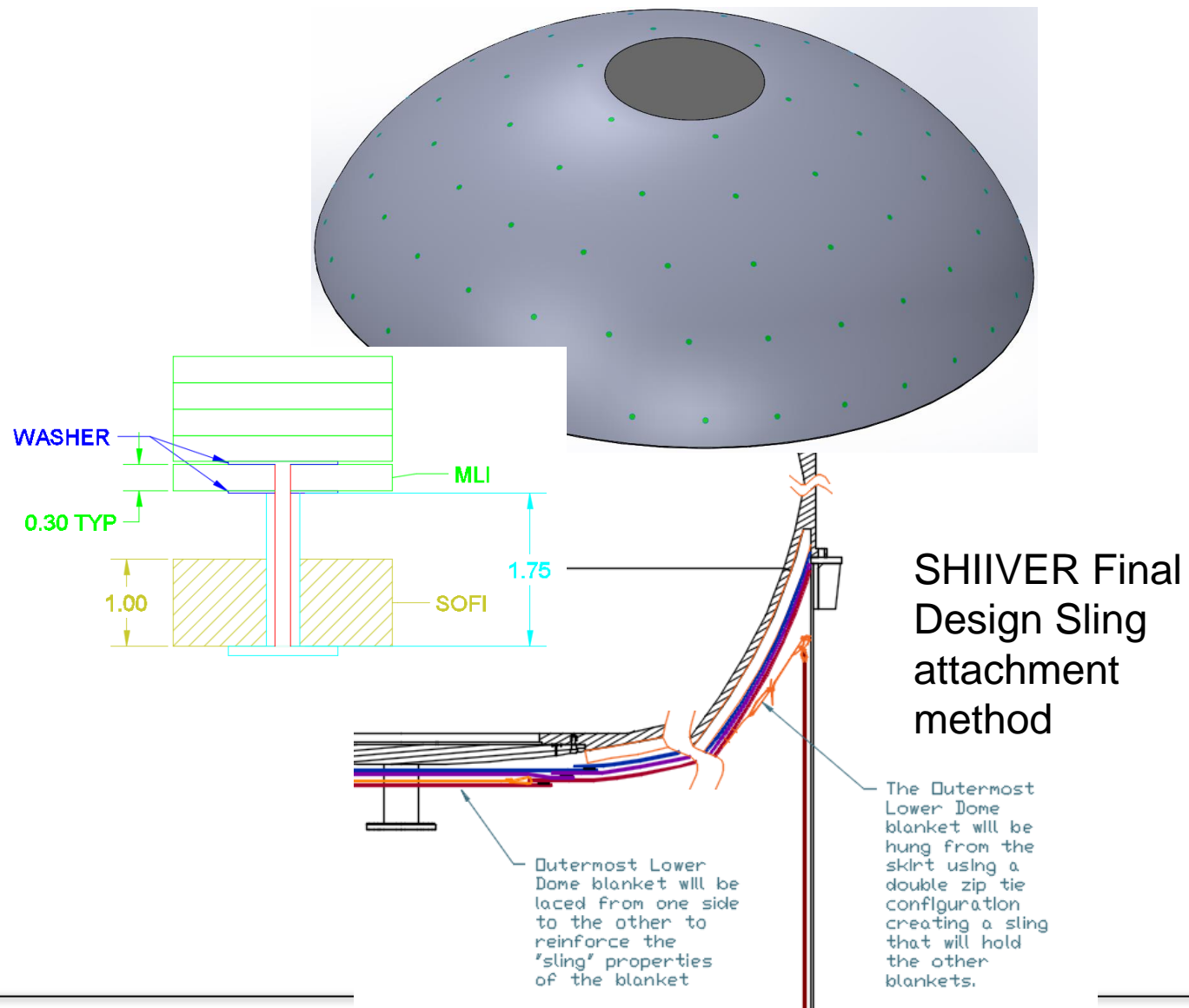


SHIIVER standoffs



- During the design of the SHIIVER MLI blankets, initially ClickBond posts were assumed to be used.
- Performed evacuation analysis
 - Average 5 Torr delta pressure across the blanket
 - 42 lb capacity per ClickBond
 - ~3000 lb force for SHIIVER
 - 90 ClickBonds for SHIIVER
 - ~13k lb force for 8.4 m upper stage
 - 397 ClickBonds – not reasonable

SHIIVER Top Dome Click-Bond Map at PDR

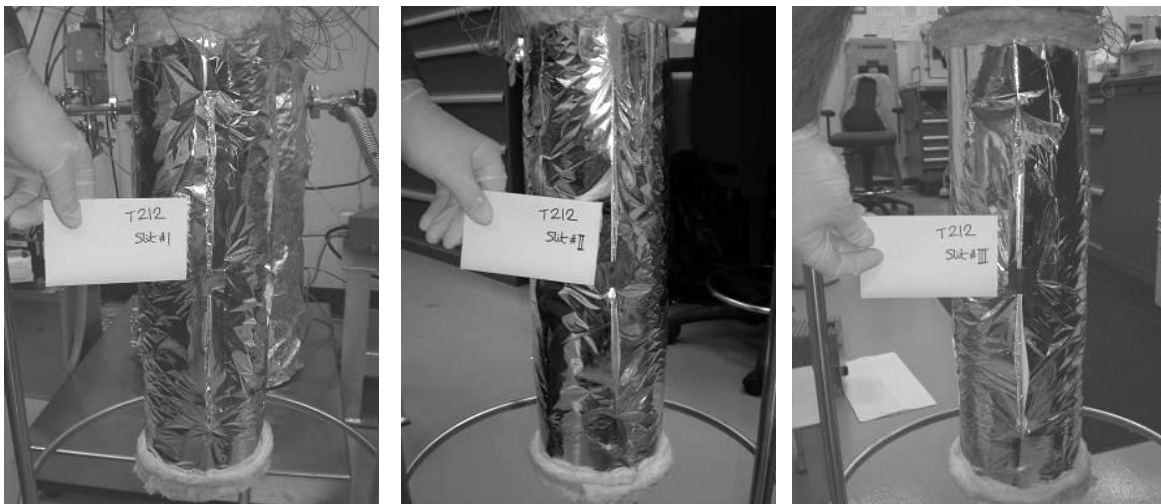




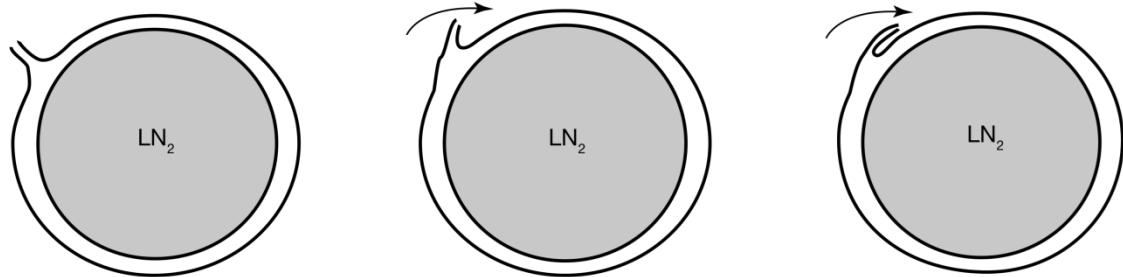
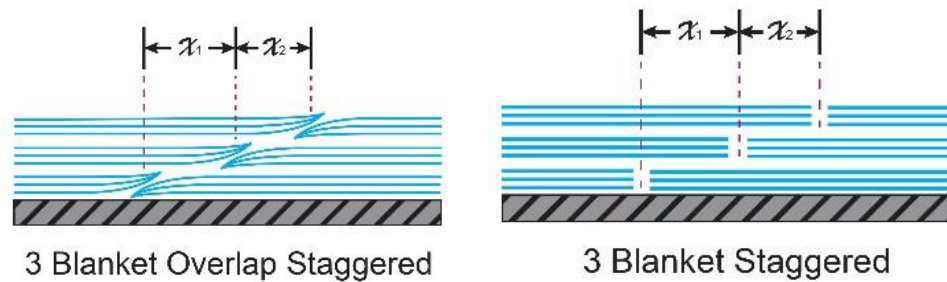
MLI Cryogenic System Heat Load Calculator for Trad Cryo MLI



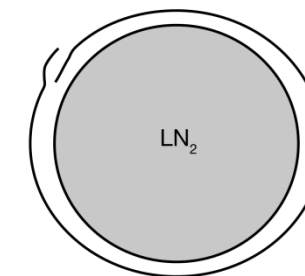
MLI Cryogenic System Heat Load Calculator										Key	Inputs	Selection	Button					
Fluid	Length Dimension	Hydrogen																
	Tank Name	meters																
		1 m tank																
Tank Diameter	Tank Length	Dome Ratio	Tank Volume	Tank Capacity	Fluid Mass	Tank Surface Area	V:SA	L/D	Tank Capacity Gallons	# Sections	Dome Surface Area	Section SA	Section Length					
1.00	1.36	1.20	0.85	0.81	57.1	4.45	0.19	1.3575	212.9	3	1.40	0.55	0.17					
Areal MLI	NewQ	K	Penetrations															
WBT	300	K	No Buffer															
CBT	20	K	Shield Emissivity															
# Layers	60	lay/cm	#	Dia (m)	dx (m)	Heat Load (W)	Note:											
Layer Density	30	lay/cm	3	0.0254	0	0.790	Hanging Points											
Vac Press	5E-06	Torr	1	0.11748	0	0.575	LH2 Fluid Interface											
C _a Coefficient	1.05E-05		1	0.11748	0	0.575	LN2 Fluid Interface											
Areal Heat Load	1.28	W/m ²	0	0	0	0.000	Total: 1.939 W											
SF	5.69	W	Penetrations Calc															
	1		Areal MLI Calc															
Total			Pins															
Blankets	5.69	W	degree	10														
Penetrations	1.94	W	radians	0.175														
Seams	0.46	W	Pin Radius	3	mm	Hole Radius	4	mm										
Pins	4.80	W	Layer Spacing	0.33333	mm	MLI Thickness	2.00	cm										
Other		W	Effective Pin Radius	4.9	mm	Pin Material	IM7-8552											
Margin	0	%	Pin Spacing	13.8	in	Pins per Area	8.10	Number Pins/m ²										
Total	12.89	W	Effective Flux	1.08	W/m ²	Total Heat	4.80	W										
										Pin Calc								
										Seams								
										Butt Seam		Overlap Seam						
										n-Layers	60	Length	10.82	m				
										Layer-Rho	30	layer/cm	Layer Density	60	lay/cm			
										Length	10.55	m	Overlap Width	2	in			
										Width	0.002	m	Qcalc	3.678	W/m ²			
										Depth	0.020	m	f	100.9				
										Stefan-Boltzman	5.67E-08	W/m ² -K ⁴	Seam Area	0.550	m ²			
										Th	300	K	Seam Load	2.023	W			
										Tc	20	K	Constant					
										qdot	0.235	W/m	Degradation	8%				
										Seam Load	2.481	W	Heat Flux	5.69				
												Seam Load	0.46	W				
										Hinckley, R.B., <i>Liquid Propellant Losses During Space Flight, Final Report</i> . NASA-CR-53336, Arthur D. Little, Inc. Cambridge, MA, 1964.								
										Structural FEA								
										T Environment	259	K	WBT	300	K	FEA Calc		
										Number of Nodes	20	CBT	20	K				
										Structure Name	#	Structure Material	Emissivity	Structure Length (m)	Structure Area (m ²)	Q _{total} (W)		
										Coupon	1	Stainless Steel 304	0.3	0.750	0.002	8.25		
												Aluminum				0.00		
												Aluminum				0.00		
												Aluminum				0.00		
										MLI Mass Tool								
										Cylindrical with Spherical Heads			Spherical					
										1.78	Cylindrical Radius, m	2.25	Spherical Radius, m					
										1.08	Cylinder Length, m	10	Section 1 MLI layers					
										0.34	Dome Height, m	0	Section 2 MLI layers					
										30.00	Section 1 MLI layers	0	Section 3 MLI layers					
										0.00	Section 2 MLI layers	10	Section 1 MLI lay dens					
										0.00	Section 3 MLI layers	15	Section 2 MLI lay dens					
										15.00	Section 1 MLI lay dens	16	Section 3 MLI lay dens					
										0.00	Section 2 MLI lay dens	0	Foam thickness, m					
										0.00	Section 3 MLI lay dens							
										OxINS	Helms	MethaneINS	HydrogenINS	Pins	Penetrations	Tank Size	Other heat Loads (He)	Other Heat ...



Butt Seams



Method of fold-over seam installation



Overlapped seams



Butt Seam Analytical Solution

- Hinckley came up with an analytical solution for the Butt Seam as shown on the right.
- Testing performed with liquid nitrogen has shown that this is a very good approximation.
- Equations are somewhat complicated and the variables may be hard to control in a real life application.
- Provides good estimate to use in initial design predictions and sensitivities.

$$\frac{\dot{Q}_{seam}}{L_{seam}} = \frac{\gamma\sigma(T_H^4 - T_C^4)}{\left(\frac{2}{\varepsilon} - 1\right)n}$$

$$\frac{\gamma}{t} = \left(\frac{2}{\varepsilon} - 1\right)n * fn\left(\frac{\delta}{t}\right)$$

$$fn\left(\frac{\delta}{t}\right) = \sqrt{1 + \varphi^2} \left(\frac{1}{3} - \frac{2\varphi^2}{3}\right) + \left(\frac{2\varphi^3}{3} - \frac{1}{3}\right) + \varphi^2 \ln\left(\frac{1 + \sqrt{1 + \varphi^2}}{\varphi}\right)$$

$$\varphi = \frac{\delta}{t}$$

Where Q is the heat leak through the seam, δ is the seam width, t is the seam depth, and L is the length of the seam.

- A series of tests on different heat load degradations was done on different cracks in a 30-layer blanket
 - Liquid nitrogen boil-off test method.
- Different ways to tape the cracks
- Different width of cracks
- Part 1 was a theoretical approach

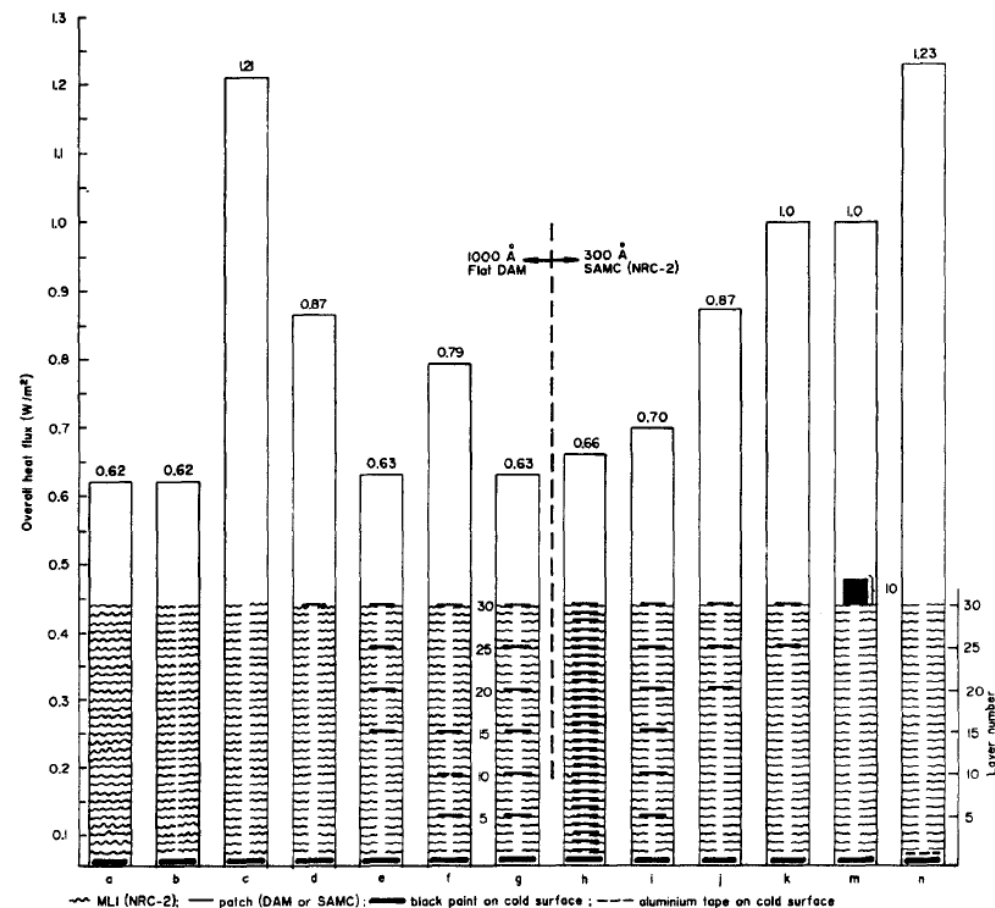


Figure 12 Graphic summary of heat flux results of patch study. a, No cracks; b, one-dimensional slits; c–m, runs 2 to 11; n, 0.09 mm aluminium tape on cold surface, no patches

Shu, Q.S., Fast, R.W., and Hart, H.L., Systematic study to reduce the effects of cracks in multilayer insulation, Part 2: experimental results, Cryogenics Vol 27, Issue 6, 1987



Overlap Seam Analytical Approach



In collaboration with Dave Frank (LMCO ret) the general thought process of treating an overlap seam as a double layer density was explored.

- **When a seam is overlapped, twice the layers of MLI are present at approximately the same thickness. Thus, the layer density is doubled.**
- **This is applied for the area of the seam using a Lockheed type equation (Modified Lockheed, NewQ, etc).**
- **It is assumed that the thermal gradients extending outside of the blanket are small in nature and do not affect the area of the blanket not a part of the seams. Thus, the rest of the blanket is treated as a nominal blanket area.**



50 Layer Test Results

- **Overlapped seams outperformed butt seams**
- **Offsetting butt seams didn't seem to provide any benefit**
 - By the time the butt seam is handled, radiation path becomes torturous
- **Minimal difference between the best and worst seams**

Test Number	Run	Q_{total} , watts	T_{avg} , K	K_{avg} , W/m/K	ΔT , K	Q_{net} , W	dQ, W	dQ, W/m
1	Overlap Seams	0.788	21.06	29.8	2.56	0.786	0.040	0.044
2	Interleaved	0.748	19.16	27.3	2.43	0.746	0.000	0.000
3	Full Butt	0.806	18.85	26.9	2.51	0.802	0.056	0.061
4	Butt 2" Offset	0.806	18.85	26.9	2.52	0.803	0.057	0.062
5	Butt 4" Offset	0.810	19.37	27.8	2.56	0.807	0.061	0.067



20 Layer Seam Results

- **Once again, Overlap seam outperformed butt seam.**
 - Minimal heat gains into system
- **Offsetting butt seams didn't provide any benefit**
- **Much bigger difference between the best and worst seaming configurations**

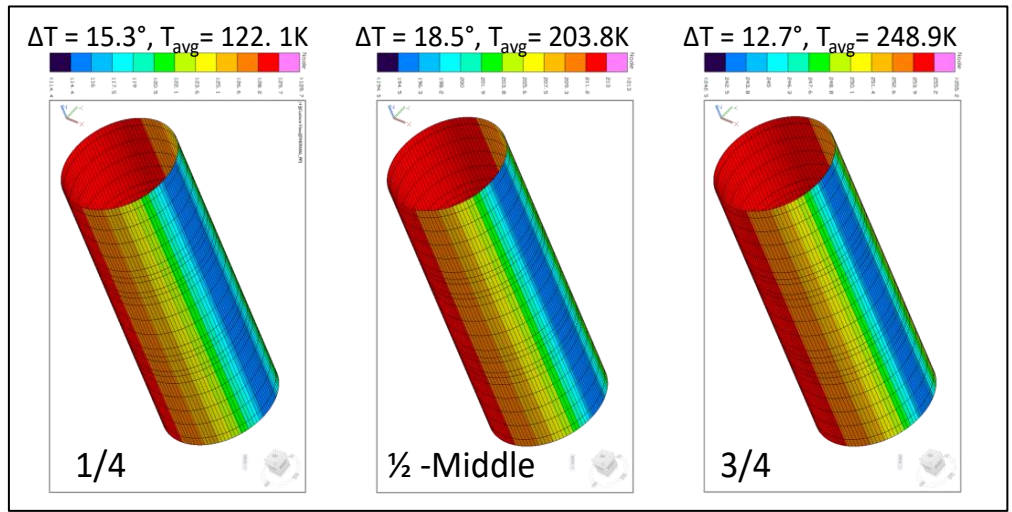
Test Number	Run	Q_{total} , watts	T_{avg} , K	K_{avg} , W/m/K	ΔT , K	Q_{net} , W	dQ, W	dQ, W/m
6	Interleaved	1.033	20.38	28.9	3.49	1.012	0	0.000
7	Overlap	1.035	18.62	26.6	3.65	1.015	0.003	0.003
8	Butt 2" Offset	1.222	17.52	25.0	4.21	1.199	0.187	0.205
9	Full Butt	1.160	17.25	24.7	4.09	1.146	0.134	0.147



SHIIVER Seams Testing



- Effect of seam measured by putting two seams into blanket
- Lockheed ATC analysis showed that two seams didn't interfere with each other thermally.
- Effect of seam approximately 0.15 W/m in this configuration
- SHIIVER MLI designed to minimize seam length on 8.4 m tank



Thermal Model by Lockheed ATC

Configuration	Q_{total}, W	Q_{net}, W	$Q_{seam}, W/m$	% change
Single Seam	0.928	0.923	0.147	
Double Seam	1.062	1.057	0.394	14.6%



Conclusions

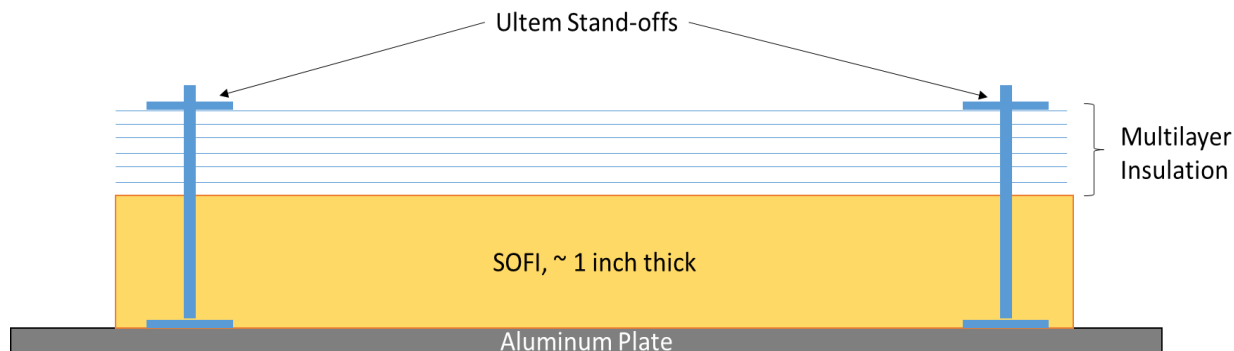


- **Measured heat loads for the nine tests conducted.**
- **Layer by layer interleaved joint showed the lowest heat leak.**
- **Overlap joint outperformed the straight and staggered butt joints.**
- **Surprisingly, staggering the butt joint did not decrease the heat load**
 - Increasing the stagger distance didn't help.
 - In fact, the test with the largest stagger was worse than straight butt joint, (although this may be due to damage incurred by repeated handling rather the joint itself).
 - Technician installed by “stitch taping” joints every ~5 layers, may have shown that stitch taping is as good as full taping
- **Even worst performing seam only 5% more heat leak than best performing seam at 50 layers**
- **There are significant differences between 20 layers and 50 layers. This shows that the impact of seams is reduced with increased numbers of layers.**

1. Johnson, W.L., and Chato, D.J., **Performance of MLI Seams between 293 K and 20 K**, *IOP Conf. Series: Materials Science and Engineering* **755** 012152, 2020.
2. Johnson, W.L., and Fesmire, J.E., **“Testing of Various Seams in MLI,”** *Advances in Cryogenic Engineering*, Vol. 55B, American Institute of Physics, Melville, NY, 2010. Pg. 905-912.

MLI Charging Test Coupons

- **6 MLI samples were installed on an aluminum plate**
 - 1-inch-thick foam substrate (material not critical, but non-conductive)
 - Each sample was approximately 8 inches by 8 inches
 - 3 coatings:
 - Single Aluminized Kapton (2 mil, referred to film 1)
 - Germanium ($\sim 1500 \text{ \AA}$) on 2 mil Kapton (film 2)
 - Indium Tin Oxide ($\sim 1.6 \text{ kohms/sq}$) on 2 mil kapton (aluminum coated underside, film 3)
 - 2 different numbers of reflective layers: 10 reflectors, 25 reflectors
 - Each sample had 3 polyetherimide (Ultem) Click-bond posts holding in place (i.e. foam was not bonded to aluminum plate)
 - 10 layer blankets each had individual grounding wires run



Vaynor, B.V., Galafaro, J.T., and Johnson, W.L., Electrostatic Testing of Multilayer Insulation for In-Space Cryogenic Vehicles, *IEEE Transactions on Plasma Science*, Vol 47, Issue 8, 2019, pp. 3810-3815.



Arcing Room Temperature Results – LEO environment



# arcs @ duration (min)	Ungrounded	Grounded	Film #1 (SAK) grounded only	Film #3 (ITO) grounded only
-200 V	0 arcs, 20 min			
-240 V	0 arcs, 30 min	0 arcs, 30 min	0 arcs, 30 min	0 arcs, 30 min
-270 V	3 arcs, 4 min			
-280 V	5 arcs, 15 min 3 arcs, 30 min*	0 arcs, 30 min	0 arcs, 30 min	0 arcs, 30 min
-320 V		0 arcs, 30 min	1 arc, 30 minutes	
-350 V	3 arcs, 30 min			
-360 V		0 arcs, 30 min		
-400 V	3 arcs, 5 min	2 arcs, 30 min	3 arcs, 1 minute	1 arc, 30 minutes
-440 V		3 arcs, 16 min		4 arcs, 30 minutes

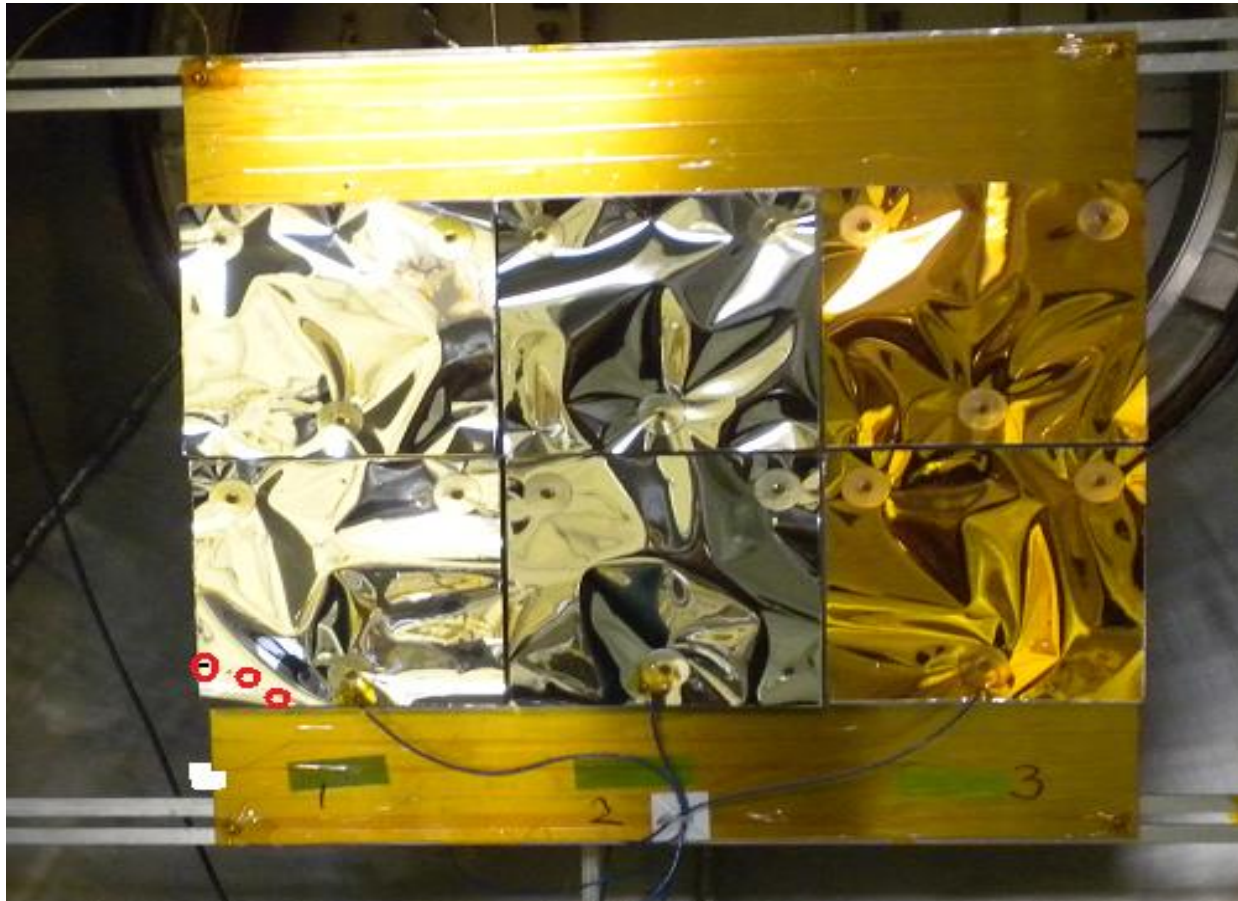
Note 1: arcs were not seen, so it is unknown which sample they occurred on.

Note 2: arcing events decreased with time, indicative of destruction of arcing sites.

*Test was re-run at higher capacitance to try to see arcing events



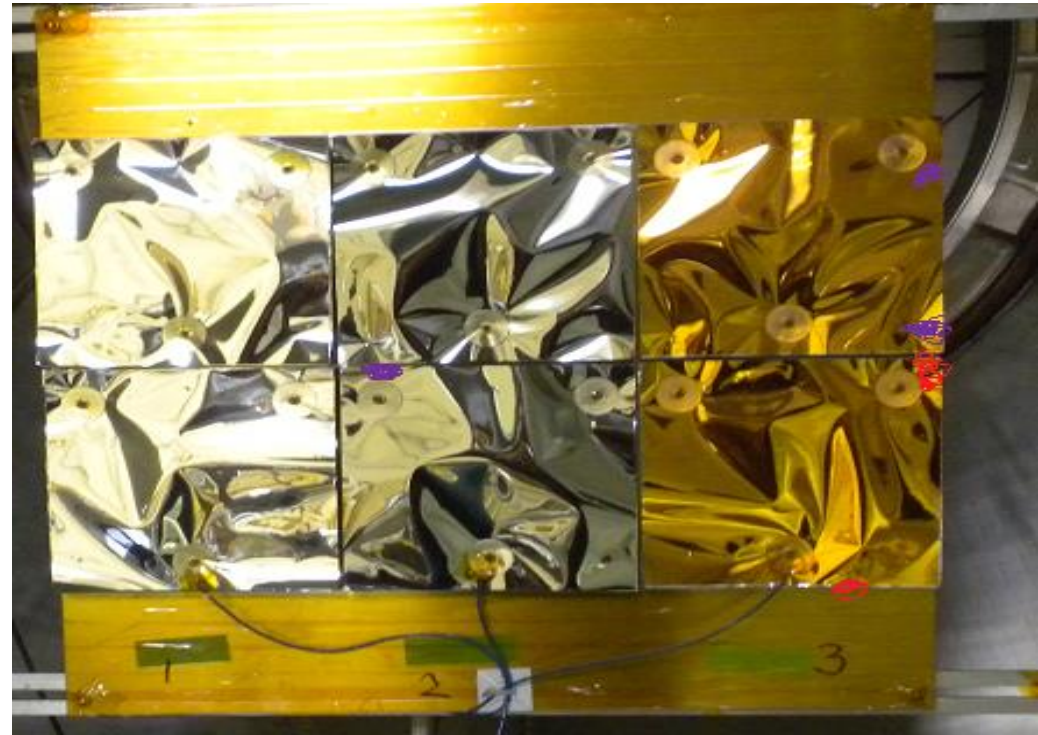
Arc locations



Film 1 (SAK) grounded only
Arcing locations shown with red circles
-400 V

Arcing Testing Results – GEO environment

# arcs @ duration (min)	Ungrounded	Films #2 and #3 grounded only	Film #1 (SAK) grounded only
2 nA/cm ²	0 arcs, 30 minutes	5 arcs, 10 s	0 arcs, 30 min
4 nA/cm ²	0 arcs, 30 minutes		0 arcs, 30 min
6 nA/cm ²			0 arcs, 30 min
8 nA/cm ²	3 arcs, 40 minutes		0 arcs, 30 min



Arcs are shown on coupon surface: red-films 2&3 grounded; violet-film 1 grounded.

Repeatability of MLI Systems

- Performed repeatability on two different sets of blanket:
 - NASA/ Sierra Lobo
 - Yetispace
- Three different temperature regimes
- All testing done on same device.

NASA Test Matrix

Test Series	Coupon #	Tc (K)	Th (K)
1	1	20	293
2	2	20	293
3	3	20	293
4	4	20	293
5	5	20	293
6	3	20	293
7	3	20	293
8	3	20	293
9	3	20	293
10	1	20	100
11	2	20	100
12	3	20	100
13	4	20	100
14	5	20	100
15	2	20	100
16	2	20	100
17	2	20	100

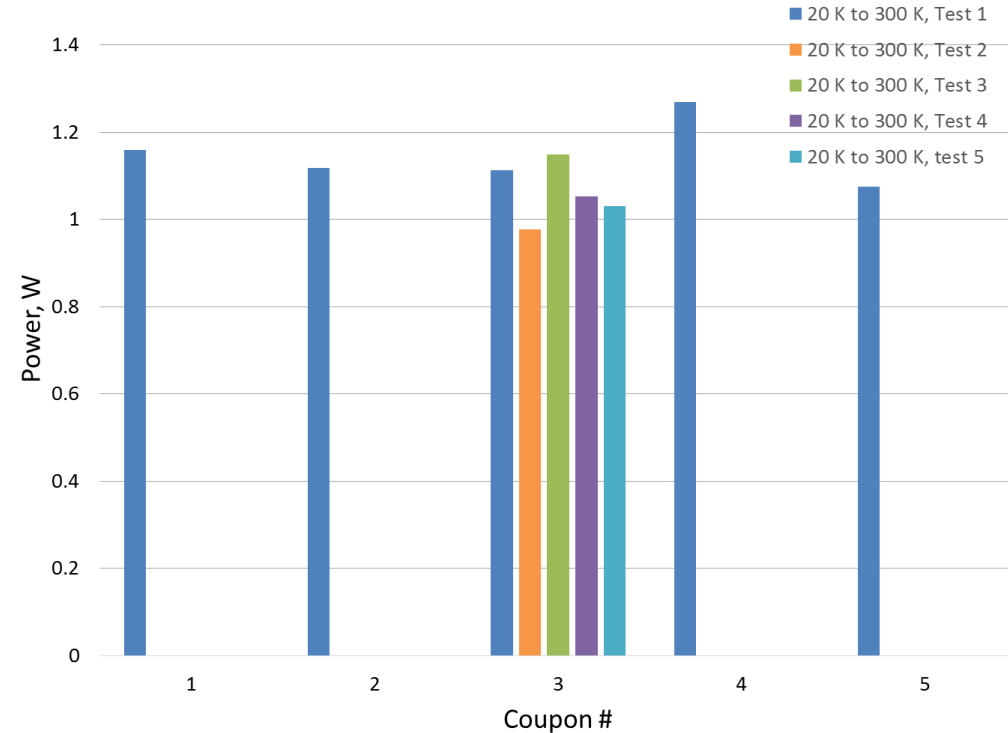
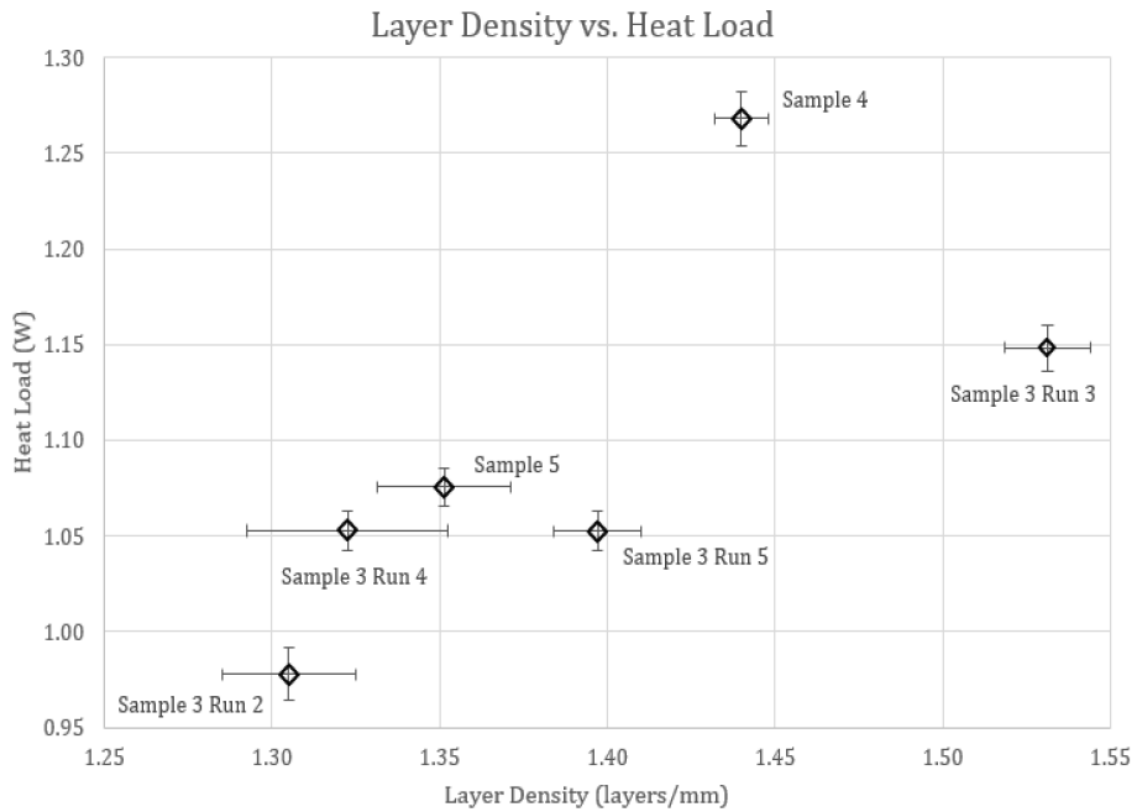
Yetispace Test Matrix

Test Series	Coupon #	Tc (K)	Th (K)
1	1	77	300
2	2	77	300
3	3	77	300
4	4	77	300
5	5	77	300
6	6	77	300
7	7	77	300
8	8	77	300
9	9	77	300
10	10	77	300

1. M. Vanderlaan, D. Stubbs, et. al. “Repeatability Measurements of Apparent Thermal Conductivity of Multi-Layer Insulation (MLI)” *IOP Conf. Series: Materials Science and Engineering* 278 012195, 2017
2. Johnson, W.L., Vanderlaan, M., et. al. Repeatability of Cryogenic Multilayer Insulation, *IOP Conf. Series: Materials Science and Engineering* 278 012196, 2017.
3. eCryo-RPT-0130



NASA coupon 300 K Test Results



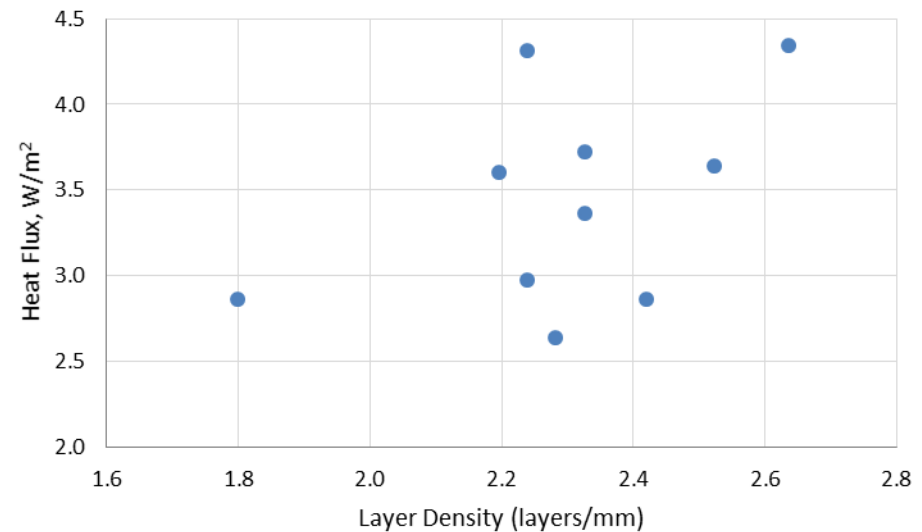
- Repeatability of single coupon installed 5 times similar to that of 5 different coupons installed:
 - Repeatability driven by installation, not coupon variability
- Small trend of performance with layer density.
- Nominally 25 reflective layers.



Yetinspace Coupon 293 K Test Data



- No discernable trends with layer density
- 10 different coupons
 - Built in two different sets of 5 blankets
 - All materials from the same lots
- Nominally 11 reflective layers
- Less repeatable than NASA coupons
 - Repeatability driven by layer count.

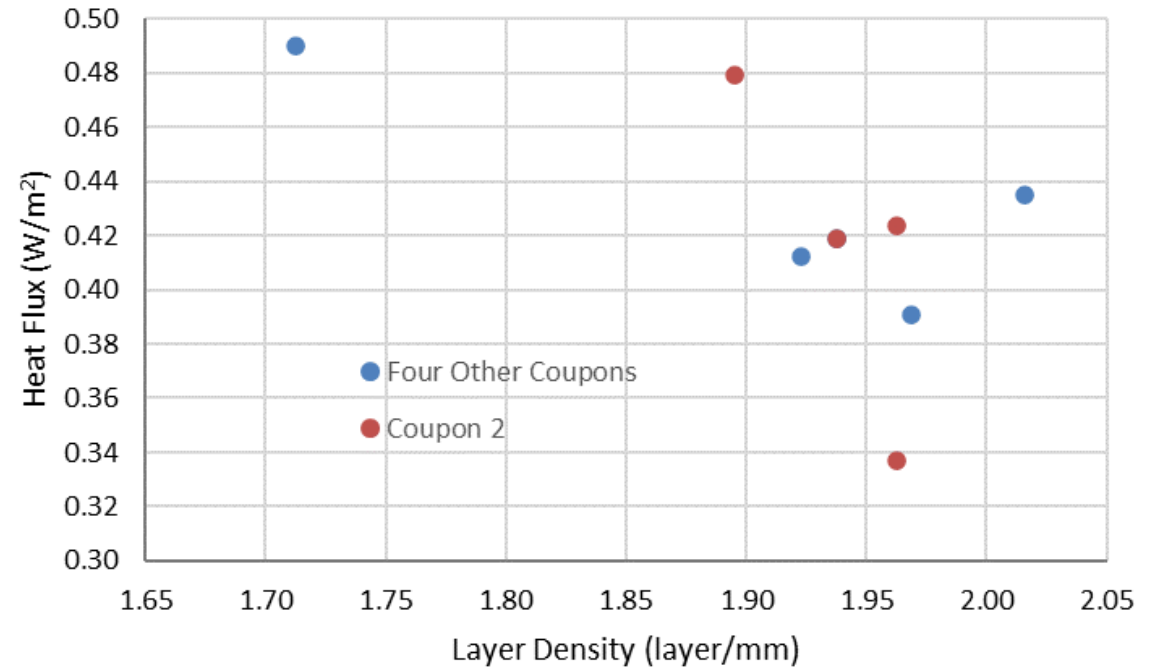




NASA coupon 100 K Test Results



- Again, no heat flux trends with layer density
- Significantly lower repeatability:
 - Multiple different installers
 - Less experience with installers





Repeatability Results



Test Series	Mean, W	Min, W	Max, W	St. Dev, W	Range, W	Uncertainty
20 K to 300 K, All Five	1.15	1.08	1.27	0.066	0.19	+/-8.4%
20 K to 300 K, Coupon 3	1.06	0.98	1.15	0.061	0.17	+/-8.0%
77K to 293K, First Five	2.40	2.05	2.80	0.27	0.75	+/- 15.6%
77 K to 293 K, Second Five	2.93	2.22	3.36	0.42	1.14	+/- 19.5%
77 K to 293 K, All ten	2.66	2.05	3.36	0.44	1.31	+/- 24.6%
20 K to 115 K, All Five	0.35	0.31	0.40	0.028	0.083	+/- 12.0%
20 K to 115 K, Coupon 2	0.33	0.27	0.39	0.041	0.115	+/- 17.2%

Evaluated per ASTM E-2586, Standard Practice for Calculating and Using Basic Statistics, 2014



Repeatability Significance



Test Series	Mean Standard Error, W	Mean SE as Percent of Mean	Calculated St. Dev, W	St. Dev Standard Error, W	St. Dev Calc – Meas, W	St. Error Greater?
20 K to 300 K, All Five	0.017	1.2%	0.083	0.023	0.017	YES
20 K to 300 K, Coupon 3	0.015	1.1%	0.074	0.021	0.013	YES
77K to 293K, First Five	0.064	2.7%	0.322	0.092	0.053	YES
77 K to 293 K, Second Five	0.098	3.3%	0.490	0.143	0.071	YES
77 K to 293 K, All ten	0.044	1.6%	0.426	0.102	-0.015	YES
20 K to 115 K, All Five	5.6×10^{-3}	1.6%	0.036	0.0095	0.0077	YES
20 K to 115 K, Coupon 2	8.2×10^{-3}	2.5%	0.056	0.016	0.015	YES

Evaluated per ASTM E-2586, Standard Practice for Calculating and Using Basic Statistics, 2014



MLI Cryogenic System Heat Load Calculator for Trad Cryo MLI



MLI Cryogenic System Heat Load Calculator										Key	Inputs	Selection	Button																																								
Fluid Length Dimension Tank Name		Hydrogen meters 1 m tank																																																			
Tank Diameter	Tank Length	Dome Ratio	Tank Volume	Tank Capacity	Fluid Mass	Tank Surface Area	V:SA	L/D	Tank Capacity Gallons	# Sections	Dome Surface Area	Section SA	Section Length																																								
1.00	1.36	1.20	0.85	0.81	57.1	4.45	0.19	1.3575	212.9	3	1.40	0.55	0.17																																								
Areal MLI WBT: 300 CBT: 20 # Layers: 60 Layer Density: 30 Vac Press: 5E-06 C _a Coefficient: 1.05E-05 Areal Heat Load: 5.69 SF: 1		Dome Ratio: K lay/cm Torr W/m ² W		Areal MLI Calc		Penetrations No Buffer Shield Emissivity: 0.03 # Dia (m) dx (m) Heat Load (W) Note: 3 0.0254 0 0.790 Hanging Points 1 0.11748 0 0.575 LH2 Fluid Interface 1 0.11748 0 0.575 LN2 Fluid Interface 0 0 0 0.000 Total: 1.939 W Penetrations Calc				Seams Butt Seam Overlap Seam n-Layers: 60 Length: 10.82 m Layer-Rho: 30 layer/cm Layer Density: 60 lay/cm Length: 10.55 m Overlap Width: 2 in Width: 0.002 m Qcalc: 3.678 W/m ² Depth: 0.020 m f: 100.9 Seam Area: 0.550 m ² Stefan-Boltzman: 5.67E-08 W/m ² -K ⁴ Seam Load: 2.023 W Th: 300 K Tc: 20 K qdot: 0.235 W/m Seam Load: 2.481 W Constant Degradation: 8% Heat Flux: 5.69 Seam Load: 0.46 W																																											
Total Blankets: 5.69 W Penetrations: 1.94 W Seams: 0.46 W Pins: 4.80 W Other: 0 W Margin: 0 % Total : 12.89 W		Overlap Seam		Pins degree: 10 radians: 0.175 Pin Radius: 3 mm Hole Radius: 4 mm Layer Spacing: 0.33333 mm MLI Thickness: 2.00 cm Effective Pin Radius: 4.9 mm Pin Material: IM7-8552 Pin Spacing: 13.8 in Pins per Area: 8.10 Number Pins/m ² Effective Flux: 1.08 W/m ² Total Heat: 4.80 W Pin Calc				Hinckley, R.B., <i>Liquid Propellant Losses During Space Flight, Final Report</i> . NASA-CR-53336, Arthur D. Little, Inc., Cambridge, MA, 1964. Structural FEA T Environment: 259 K WBT: 300 K Number of Nodes: 20 CBT: 20 K FEA Calc Structure Name # Structure Material Emissivity Structure Length (m) Structure Area (m ²) Q _{total} (W) Coupon 1 Stainless Steel 304 0.3 0.750 0.002 8.25 Aluminum 0.00 Aluminum 0.00 Aluminum 0.00																																													
MLI Mass Tool <table border="1"> <thead> <tr> <th colspan="2">Cylindrical with Spherical Heads</th> <th colspan="2">Spherical</th> </tr> </thead> <tbody> <tr> <td>1.78</td> <td>Cylindrical Radius, m</td> <td>2.25</td> <td>Spherical Radius, m</td> </tr> <tr> <td>1.08</td> <td>Cylinder Length, m</td> <td>10</td> <td>Section 1 MLI layers</td> </tr> <tr> <td>0.34</td> <td>Dome Height, m</td> <td>0</td> <td>Section 2 MLI layers</td> </tr> <tr> <td>30.00</td> <td>Section 1 MLI layers</td> <td>0</td> <td>Section 3 MLI layers</td> </tr> <tr> <td>0.00</td> <td>Section 2 MLI layers</td> <td>10</td> <td>Section 1 MLI lay dens</td> </tr> <tr> <td>0.00</td> <td>Section 3 MLI layers</td> <td>15</td> <td>Section 2 MLI lay dens</td> </tr> <tr> <td>15.00</td> <td>Section 1 MLI lay dens</td> <td>16</td> <td>Section 3 MLI lay dens</td> </tr> <tr> <td>0.00</td> <td>Section 2 MLI lay dens</td> <td>0</td> <td>Foam thickness, m</td> </tr> <tr> <td>0.00</td> <td>Section 3 MLI lay dens</td> <td></td> <td></td> </tr> </tbody> </table>														Cylindrical with Spherical Heads		Spherical		1.78	Cylindrical Radius, m	2.25	Spherical Radius, m	1.08	Cylinder Length, m	10	Section 1 MLI layers	0.34	Dome Height, m	0	Section 2 MLI layers	30.00	Section 1 MLI layers	0	Section 3 MLI layers	0.00	Section 2 MLI layers	10	Section 1 MLI lay dens	0.00	Section 3 MLI layers	15	Section 2 MLI lay dens	15.00	Section 1 MLI lay dens	16	Section 3 MLI lay dens	0.00	Section 2 MLI lay dens	0	Foam thickness, m	0.00	Section 3 MLI lay dens		
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OxINS	Helms	MethaneINS	HydrogenINS	Pins	Penetrations	Tank Size	Other heat Loads (He)	Other Heat ...																																													



Quest xMLI Products





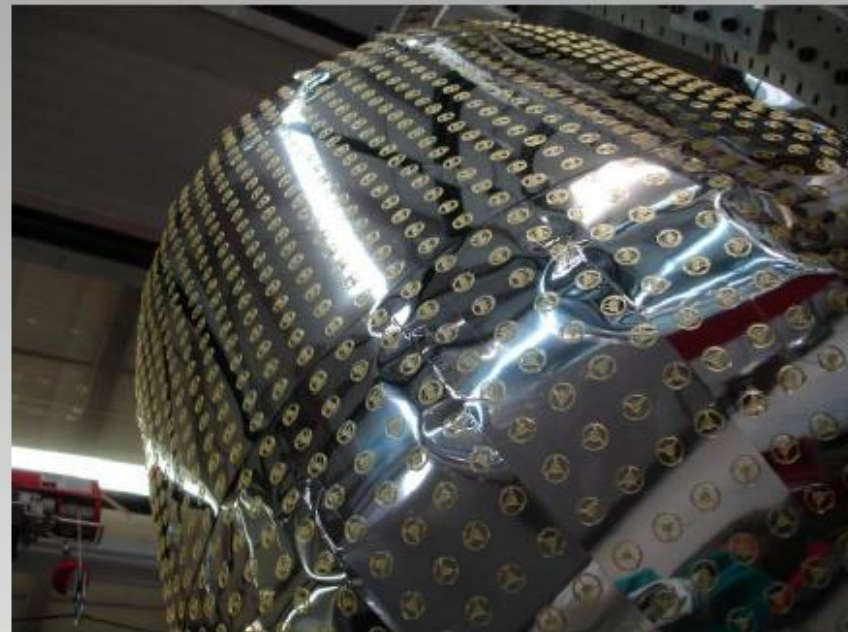
A summary of the History of the xMLI family

- **Integrated MLI was the first of the xMLI family developed**
 - Funded through an SBIR contract in 2006
 - Joint effort between Quest Product Development (spun off Quest Thermal Group) and Ball Aerospace
 - Phase 2 awarded in 2006, multiple Phase 3 awards since
- **Load Responsive MLI (LR-MLI) was funded as a Phase 1 SBIR in 2008**
 - Developed to allow insulations to structurally carry vacuum jackets for science dewars.
 - Phase 2 subsequently awarded with multiple Phase 3 awards
- **Wrapped MLI was funded as a Phase 1 SBIR in 2009**
 - Insulate vacuum jacketed lines
 - Phase 2 was awarded – no NASA work past this contract
- **Multiple other Phase 1 and 2 SBIRs were awarded to Quest between 2010 and the present**
 - Complex shapes
 - Vapor Cooled Shield -MLI
 - Multi-Environment MLI
 - MMOD-IMLI
 - Advanced Cooled Shield – IMLI
 - Variable Conductance Radiators
 - Launch Vehicle IMLI
 - Vapor Cooled Structure MLI
 - Etc
- **Load Bearing MLI developed in response to the Self-Supporting MLI solicitation that lead to the 2nd round of Reduced Boil-off Testing as a part of CPST in 2012.**



IMLI Physical Implementation

- For IMLI, the tripod spacers are on an approximately 2-inch square grid
- For LR-MLI, the dual-tripod spacers are on an approximately 1-inch square grid (holds 15 psid)
- For LB-MLI/SS-MLI, the dual-tripod spacers are on an approximately 2-inch square grid
 - Holds much less than 15 psid





IMLI / LB-MLI Testing – Bulk Blankets



- There are on the order of 30 test data points on bulk blankets that have been published outside of Quest in-house testing.
- Most of these have been done by NASA either at KSC on Cryostat-100 or GRC during either the Reduced Boil-off 2 testing or on the CoMPACT calorimeter.
 - MSFC VATA-2 testing was also conducted in parallel to the RBO-2 testing, but the data was not cleaned up or evaluated. If done so, this could provide interesting data points in the mid warm boundary temperature range.
- Tank testing has been done on RBO-2 and VATA-2
- The presenter has been a part of all of these tests.

Non-Quest Testing

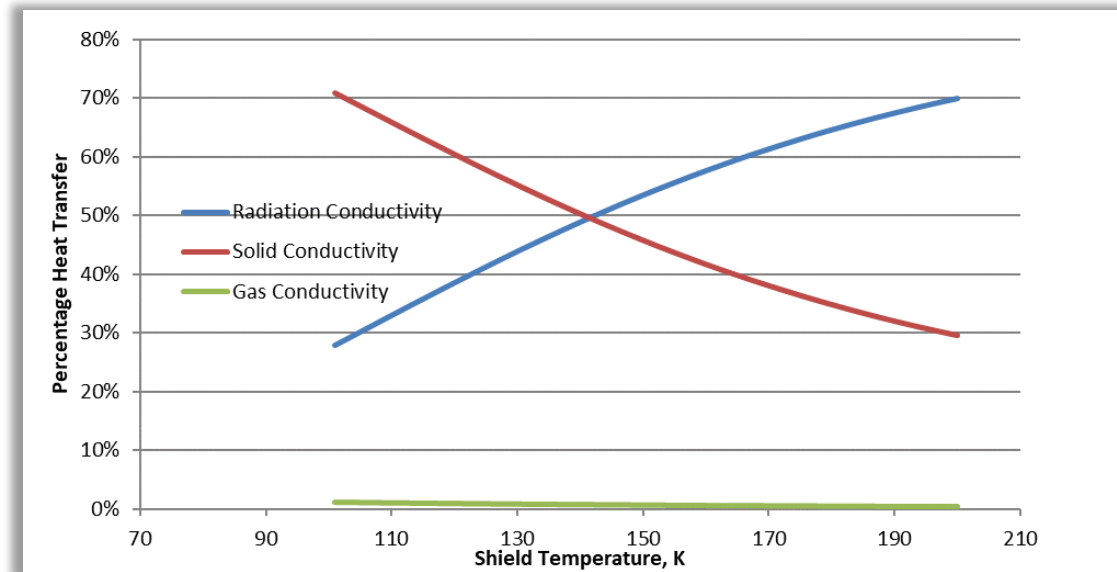
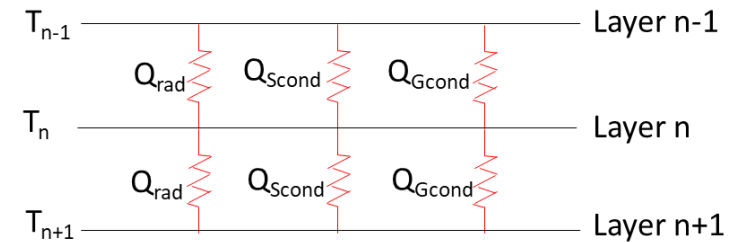
Test Site	layers	Tc, K	Th, K	Measured Q, W/m ²
Ball	10	76	296	0.95
KSC	20	77	292	0.41
KSC	20	77	305	0.567
Ball	3	76	296	3.62
KSC	9	78	293	0.924
	9	78	325	1.358
	9	78	316	1.232
KSC	5	78	293	1.772
	5	78	305	1.989
	5	78	325	2.609
KSC	19	78	293	0.545
	19	78	305	0.768
	19	78	327.8	0.849
FSU	4	20	85	0.18
	9	20	85	0.13
KSC	10	77	181	0.41
	10	77	178	0.395
	10	77	190	0.376
	10	77	194	0.552
	12	77	219	0.542
	16	77	261	0.868
	16	77	268	0.868
	20	77	265	0.828
	14	77	254	0.814
GRC RBO	19	25	80	0.085
	19	29.1	80	0.0788
	19	32	182	0.219
	19	25	87	0.097
	19	23	253	0.453
GRC	10	20	74	0.186
Calorimeter	10	20	90.7	0.206
	10	20	72.3	0.168
	10	20	90.6	0.185
	20	20	75.5	0.158
	20	20	90.6	0.169
	20	20	90.6	0.158
	20	20	75.4	0.146

LB-MLI in green



IMLI / LB-MLI / LR-MLI Analysis

- **NASA and Quest have independently developed similar analysis approaches**
- **Both employ a “layer by layer” model approach where conduction, radiation, and convection are accounted for between each layer.**
 - The temperature of each layer is iterated upon until uniform heat flux achieved through each layer.
 - Conductors modeled as an A/L and number of spacers per square meter.
- **Results are generally good for IMLI**
- **LB-MLI / LR-MLI harder due to trying to model if any contact is made between center tripod and previous layer (much higher A/L).**
 - If no contact, IMLI model works for these too





IMLI / LB-MLI / LR-MLI flights



- **IMLI has flown or is getting ready to fly on multiple missions:**
 - Green Propellant Infusion Mission (Ball Aerospace prime) as a secondary mission demonstration
 - Robotic Refueling Mission 3 on the receiver dewar
 - Note that no methane transfer occurred on this mission, though ground test data indicated the IMLI functioned as planned.
 - Lucy (Jupiter Trojan Asteroids)
 - On L’Ralph Instrument Package
- **Planned for several other missions:**
 - Near Earth Object Surveyor (2027)
 - The Lunar Environment Monitoring Station (LEMS)
 - Roman Space Telescope (mid-2020s)





IMLI /LB-MLI Other Test Data



Test data has been gathered on:

- Seaming technic: result showed no change between 1 and 2 seams
- Perforations: For RBO-2/VATA-2, Quest perforated their blanket at less than 0.01% open area
 - 0.25-inch diameter holes on 24 inch spacing
 - < 5% change in performance degradation during calorimeter testing, within the uncertainty of the test hardware.
 - Successfully survived all testing, even with SOFI outgassing
- Penetrations:
 - Small penetrations – wires, etc. characterized similar to traditional cryogenic MLI
 - Medium penetrations tested on 0.25”, 0.5” diameter penetrations
 - Vacuum gaps and cryolite filler
 - 6 tests, data analysis partially performed but never published
 - Was not modeled like traditional MLI was
- Vibration survivability
 - Qualification testing for GPIM
 - As a part of RBO-2/VATA-2
- Acoustic survivability
 - As a part of RBO-2/VATA-2
- Performance as a function of pressure
- Rapid depressurization (see RBO-2)

TABLE 3. Comparison of 10 layer blanket degradation due to seams and perforation.

Heat Flux (W/m ²)	WBT of 293 K	WBT of 325 K
Control	0.92	1.36
Perforated (0.01%)	0.98	1.33
Two seams	0.92	1.21



Conclusions



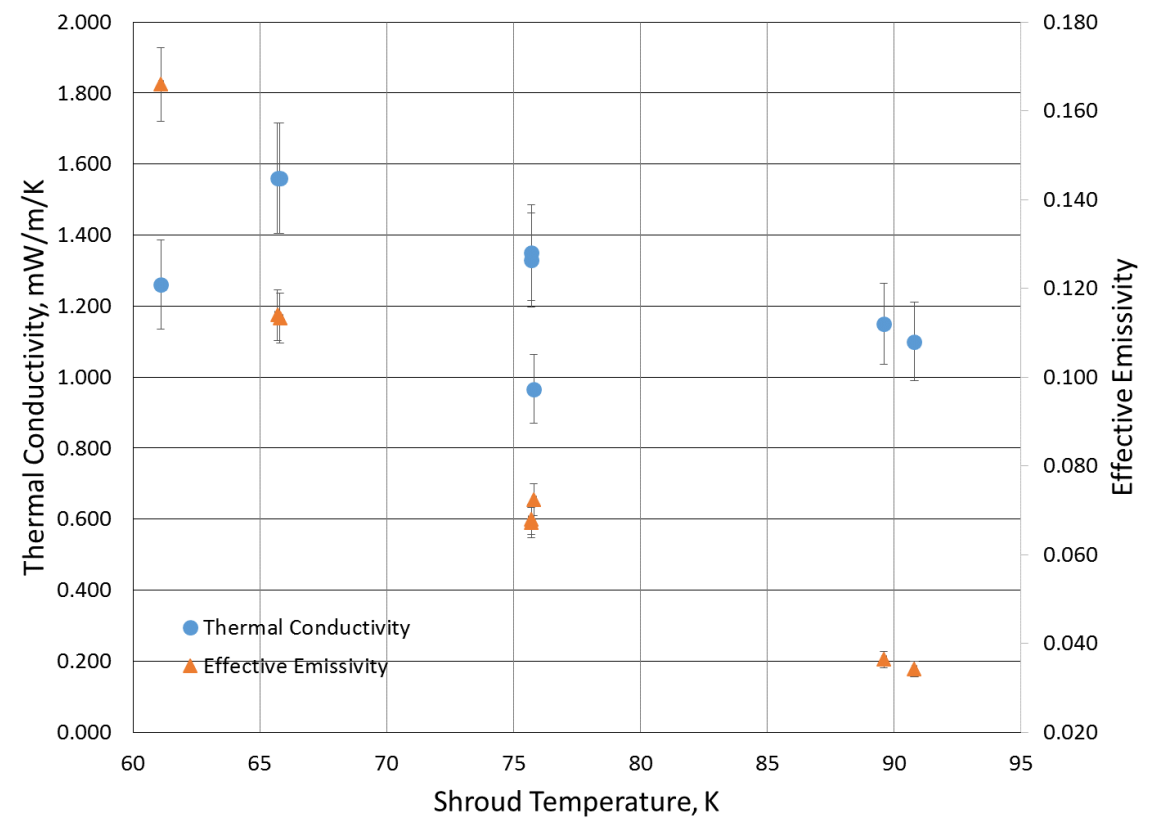
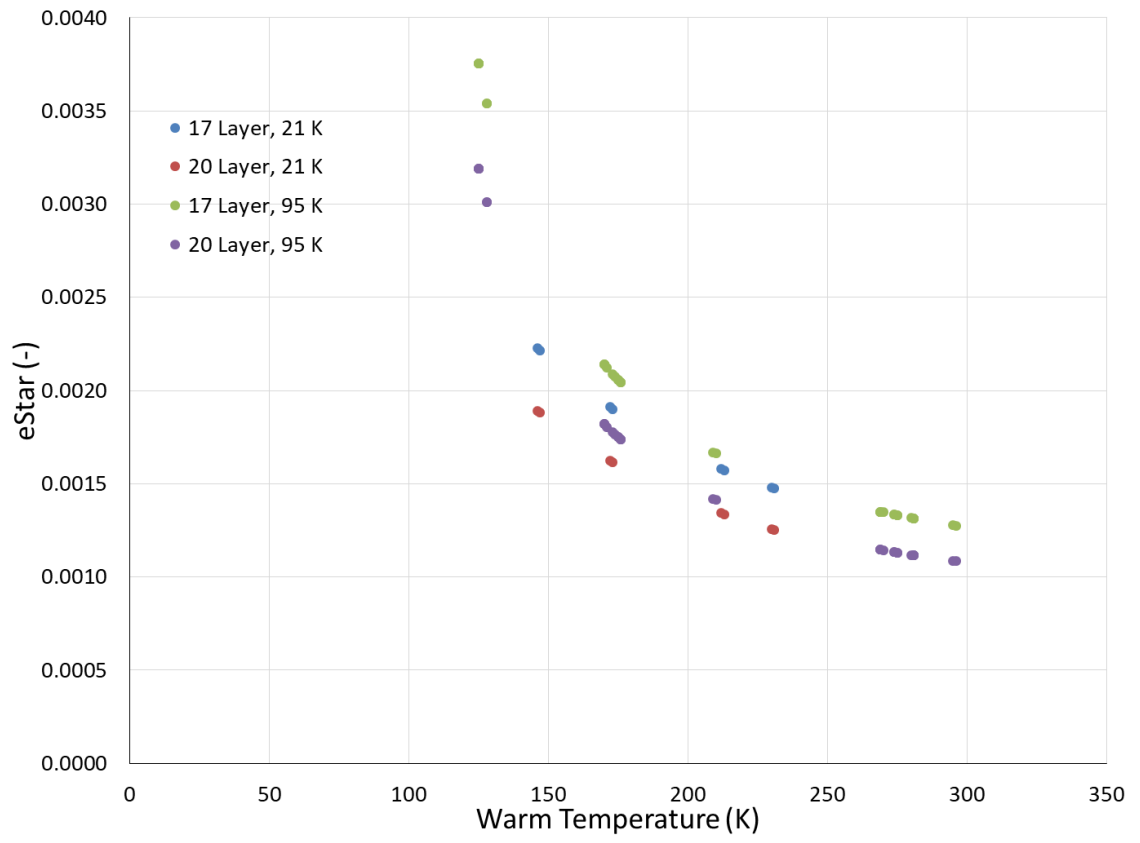


Other Somewhat Related Lessons Learned

- **How do you specify MLI**
 - Layer count?
 - Performance (effective emissivity, heat flux, etc.)?
 - How do you verify the specification
- **How do you normalize MLI performance? Why?**
- **If you use data – does that data describe how it was tested?**
 - How do you test MLI?
- **Read NASA TP- 20205008233, Appendix J for thorough discussion of lessons learned from SHIVER (how to apply high performance MLI to a large tank)**



Estar with Temp





Summary



- **MLI performance is driven by many variables, all of which must be controlled in some manner.**
- **The expected performance of a blanket must be anchored in some manner to an analytical justification:**
 - How is the system being installed?
 - How is the system being fabricated?
 - What are the driving requirements and environments to consider?
- **The aerospace industry has a long history behind current MLI development that is often not well remembered.**
- **Much of the data that has been generated more recently is considered company proprietary data.**



Questions?





LIQUID OXYGEN ZERO BOIL-OFF TEST MLI PERFORMANCE



MLI Heat Load

- Tank Surface area: 6.18 m²
- MLI inner SA: 6.23 m²
- MLI thickness: 1.25 inch
- MLI outer SA: 6.87 m²
- Mean Insulation SA: 6.57 m²
- MLI temperature gradient essentially constant through all tests at 220 K– indicates nearly identical heat load

Warm Boundary Temperature (K)	220	300
Cold Boundary Temperature (K)	95.4	78.6
MLI/remainder Heat Load (W)	2.6	4.8
Heat Flux (W/m ²)	0.38	0.74
Effective emissivity, ϵ^*	0.0029	0.0016

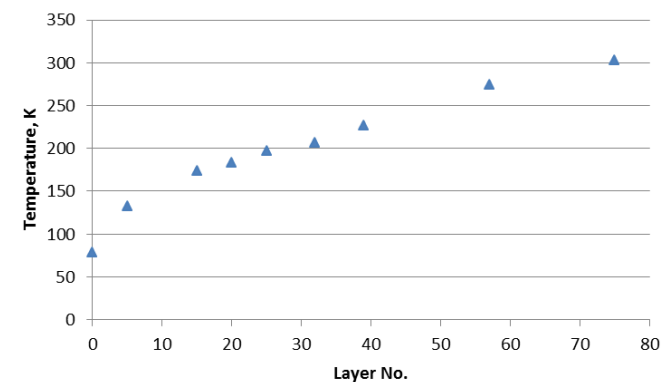
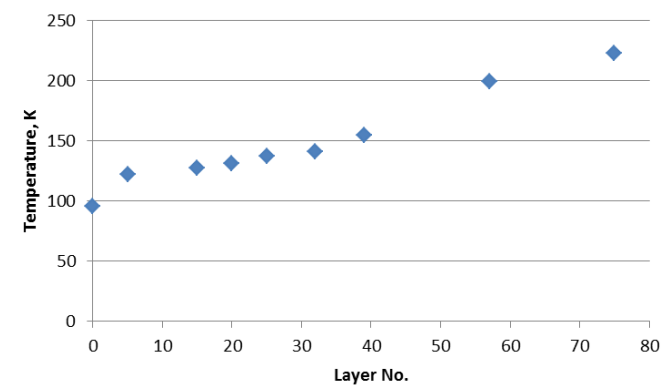


MLI Blankets



- **MLI was split into 2 blankets**
- **38 layers in each blanket**
 - TC-53 in between two blankets
 - Gives boundary temperature
- **Intermediate Temperature:**
 - 154 K @ 220 K WBT
 - 227 K @ 300 K WBT
- **Effective emissivities (ϵ^*):**
 - Outer blanket:
 - 0.0037 @ 220 K WBT
 - 0.0023 @ 300 K WBT
 - Inner blanket:
 - 0.014 @ 220 K WBT
 - 0.0050 @ 300 K WBT
 - Bunching of the inner blanket may have caused issue
 - Appears to be a temperature/emissivity issues
 - Data later shows that this may be captured in existing models

Sensor	Layer (from bottom)	Temperatures	
		220 K WBT	300 K WBT
Shroud		218.8	298.4
TC-51	75	222.8	302.8
TC-52	57	199.1	274.0
TC-53	39	154.2	226.9
TC-59	32	140.8	206.3
TC-58	25	137.3	197.6
TC-57	20	131.1	183.0
TC-56	15	127.4	174.0
TC-55	10	Off Scale High	
TC-54	5	121.6	132.8
CBT	0	95.4	78.6





Conventional MLI Heat Leak Calculations

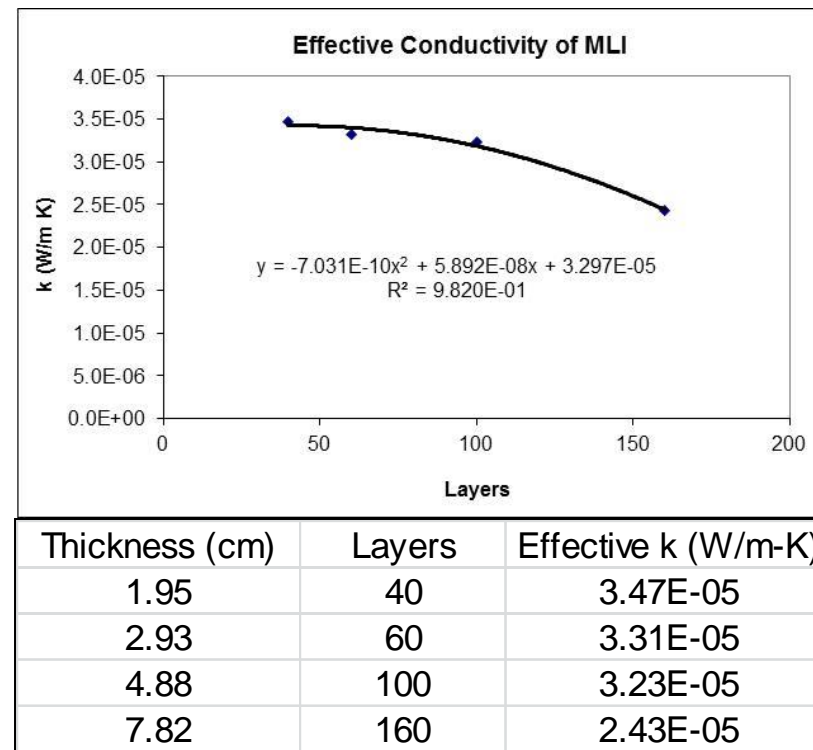
- Number of Layers: 75
- Blanket Thickness: 3.8 cm (1.5 inches)
- Layer Density: 20 lay/cm
- Perforations: Only in outer/inner layer of sub-blankets
- Used four different methods to calculate areage heat loads:
 - Thermal conductivity from Stochl¹
Note: The effective conductivity model is intended to use for calculation heat leak through MLI blanket (not temperature profile)
 - NewQ Equation²
 - Lockheed Equation³
 - Modified Lockheed Equation⁴

¹Stochl, NASA TN-D-7659

²Johnson, W.L. ***Thermal Performance of Cryogenic Multilayer Insulation at Various Layer Spacings***, Master's Thesis, University of Central Florida, Dec. 2010.

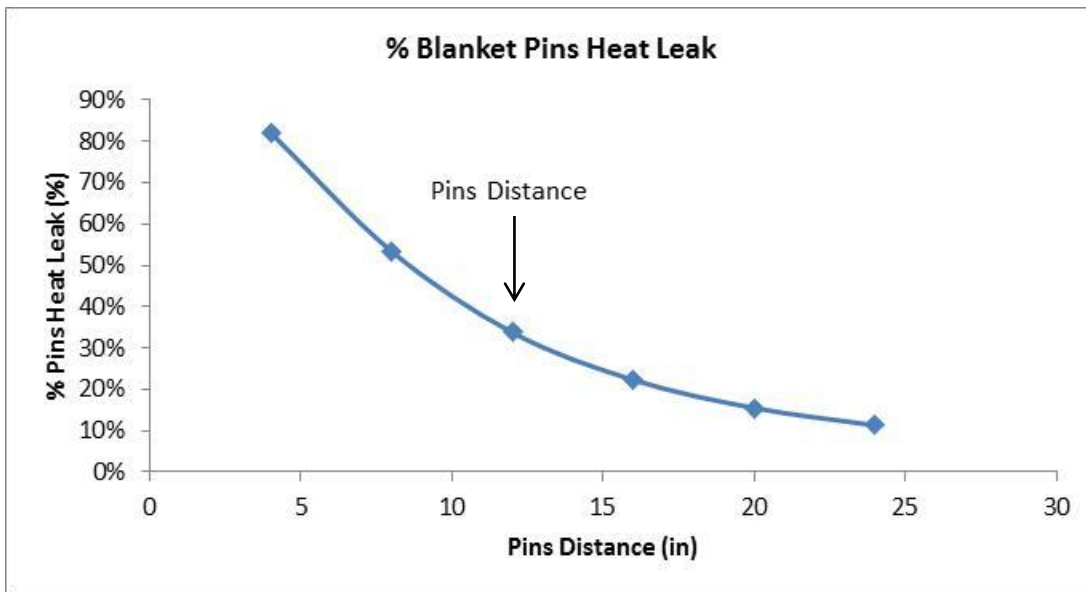
³Eq 4-56 from NASA CR-134477

⁴Eq 13 from NASA TM-2004-213175





LZBO Tank MLI Heat Leak



- LZBO tank MLI is reinforced using 0.080" nylon blanket pins for structure support
- Pin spacing a trade between structural integrity and thermal performance.
- Design distance between pins is 12" which results 33.7% additional heat leak compared to conventional MLI or 10% overall additional heat leak into the tank

LZBO Tank Conventional Blanket Heat Leak Break-up

Th = 250 K, Tc = 90 K, 75 layers, 0.080" diam, Nylon Pin, 1.1" length

Pin Distance (in)	Blanket (W/m ²)	Pins/m ² (#/m ²)	Pins (W/m ²)	Total (W/m ²)	% Pins Heat Leak (%)
4	0.146	97	0.668	0.814	82.1%
8	0.146	24	0.167	0.313	53.4%
12	0.146	11	0.074	0.220	33.7%
16	0.146	6	0.042	0.188	22.2%
20	0.146	4	0.027	0.173	15.5%
24	0.146	3	0.019	0.165	11.3%



MLI seams

- **Design (scaled from Sumner, TN D-8229)**
 - Butt joint
 - 3.2 mm gap
 - Offset with each blanket
 - 0.06 W/m
- **Check with Hinkley (CR-53336, pg II-29)**
 - 0.13 W/m for upper blanket
 - 0.03 W/m for lower blanket
 - 0.11 W/m for full blanket (if seams aligned)
- **3.5 m seam length**
- **0.39 W @ 220 K WBT**

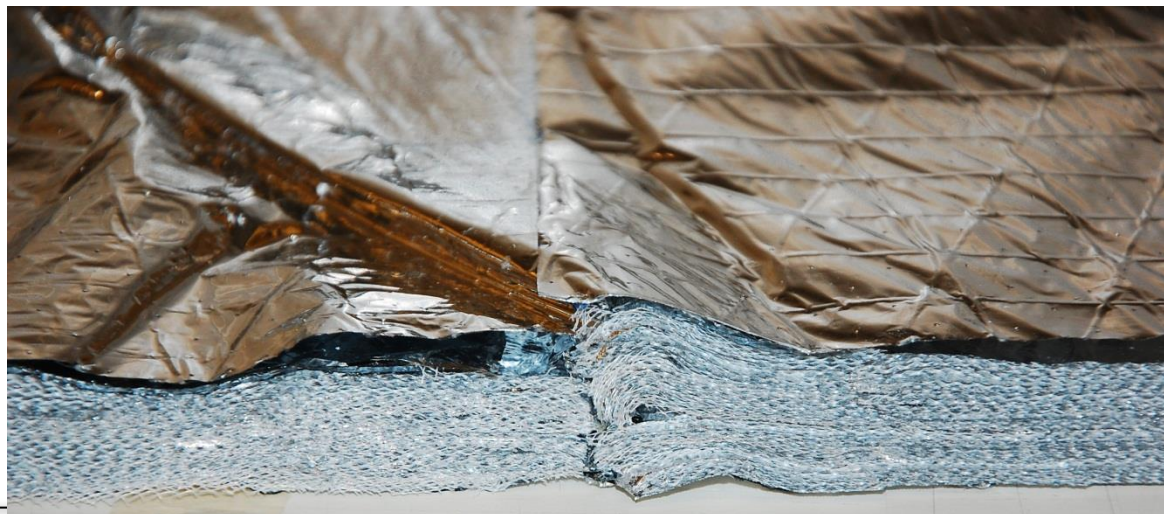
$$\dot{q} = l_s \delta_s f \sigma (T_H^4 - T_C^4)$$

where

l_s = seam length, ft

δ_s = seam depth, ft

f = dimensionless function of the butted joint to depth ratio ($f = .0112$ for 1/16 inch width)



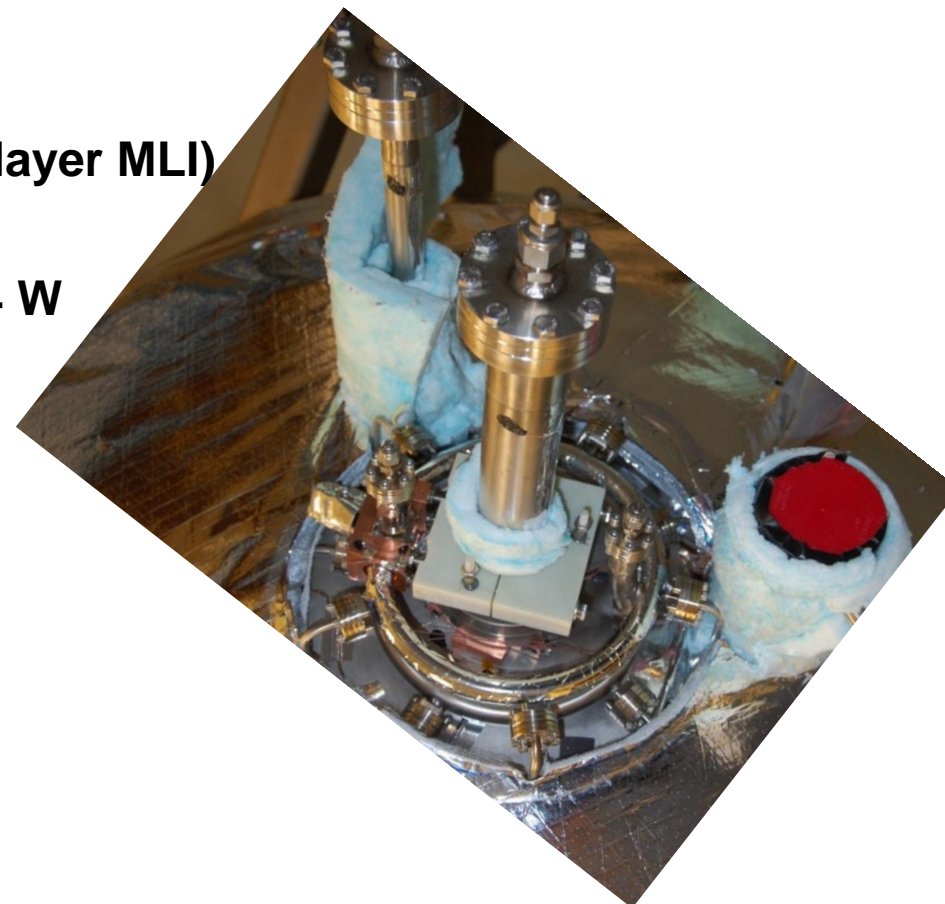


MLI Penetrations



- **Fluid/electrical penetrations isolated from MLI**
 - 0.5" (12.5 mm) thick Cryolite
 - 75 layers of MLI
- **2 inch OD vent line: 0.10 W (only 35 layer MLI)**
- **1 inch OD fill line: 0.02 W**
- **~3 inch OD instrumentation leg: 0.04 W**
- **1 inch OD struts (6): 0.09 W**
- **Total 0.25 W @ 220 K WBT**

- **NASA TP-2012-216315**



$$dq = q_{ref} \left(\frac{q_{actual}}{q'_{ref}} \right)_{\#layers} \left(\frac{q'_{ref}}{q_{ref}} \right)_{buffer\ thick} \left(\frac{q_{actual}}{q_{ref}} \right)_{diameter} \left(\frac{q_{actual}}{q_{ref}} \right)_{buffer\ thick} \left(\frac{T_h}{297} \right)^{1.56}$$



Compiled MLI Heat Loads at 220K WBT

Heat Load Method	Heat Load (W)	Total System Heat Load (W)	Ins. System Scale Factor	Blanket Scale Factor
NASA TN-D-7659 1	0.861	1.80	1.5	3.0
Lockheed Equation ³	0.657	1.59	1.7	4.0
New Q Equation ²	1.60	2.53	1.0	1.6
Modified Lockheed ⁴	1.58	2.51	1.0	1.7
Pin Heat Load*	0.298			
Seam Heat Load	0.39			
Penetrations Heat Load	0.25			
Actual Heat Load		2.6		

$$SF_{blanket} = \frac{Q_{actual}}{Q_{MLI}}$$

$$SF_{system} = \frac{Q_{actual}}{Q_{MLI} + Q_{pin} + Q_{seam} + Q_{pen}}$$

As engineering design of MLI blanket matures, accounting for components allows for more accurate MLI heat load calculation.

High layer density (20 lay/cm) caused conduction through Dacron netting to be increasingly important.



Compiled MLI Heat Loads at 300 K WBT

Heat Flux Method	Heat Load (W)	Total System Heat Load (W)	Ins. System Scale Factor	Blanket Scale Factor
NASA TN-D-7659 ¹	1.55	3.80	1.3	3.1
Lockheed Equation ³	1.66	3.91	1.2	2.9
New Q Equation ²	3.02	5.27	0.92	1.6
Modified Lockheed ⁴	3.21	5.46	0.89	1.5
Pin Heat Load*	0.53			$SF_{blanket} = \frac{Q_{actual}}{Q_{MLI}}$
Seam Heat Load	1.32			
Penetrations Heat Load	0.40		$SF_{system} = \frac{Q_{actual}}{Q_{MLI} + Q_{pin} + Q_{seam} + Q_{pen}}$	
Actual Heat Load		4.8		

As engineering design of MLI blanket matures, accounting for components allows for more accurate MLI heat load calculation.

High layer density (20 lay/cm) caused conduction through dacron netting to be increasingly important.