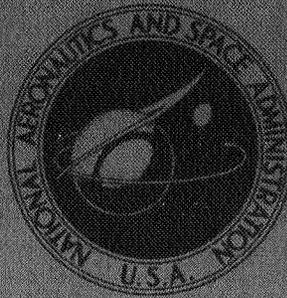


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EXPERIMENTAL LATERAL BENDING DYNAMICS  
OF THE ATLAS-CENTAUR-SURVEYOR  
LAUNCH VEHICLE

*by Robert P. Miller and Theodore F. Gerus*

*Lewis Research Center*

*Cleveland, Ohio*

NATIONAL AERONAUTICS AND SPACE ADMINISTRATION • WASHINGTON, D. C. • JULY 1969

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## ABSTRACT

A series of full-scale dynamic tests was conducted to determine the lateral bending dynamics of the Atlas-Centaur-Surveyor launch vehicle. The tests were made at several simulated flight times to obtain a good experimental definition of the vehicle dynamic response in the bending mode frequency range. Parameters measured were bending mode frequencies, damping, mode shapes, and linearity. The bending mode frequencies measured ranged from 2.59 to 31.9 Hz. Damping measured averaged about 2.0 percent of critical. Mode shapes were well defined and continuous in the lower-frequency modes but indicated discontinuities in the higher modes. The modal response was very linear over the excitation range chosen, and only small differences between pitch and yaw axes were seen. Phase relations of various response points for three modes were also measured.

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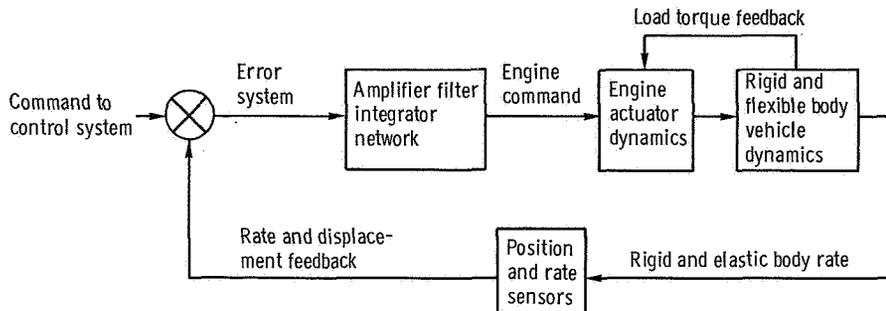
## SUMMARY

A series of full-scale dynamic tests was conducted to determine the lateral bending dynamics of the Atlas-Centaur-Surveyor launch vehicle. The tests were made at several simulated flight times to obtain a good experimental definition of the vehicle dynamic response in the bending mode frequency range. Parameters measured were bending mode frequencies, damping, mode shapes, and linearity.

The bending mode frequencies measured ranged from 2.59 to 31.9 hertz. Damping measured averaged about 2.0 percent of critical. Mode shapes were well defined and continuous in the lower-frequency modes but indicated discontinuities in the higher modes. The modal response was very linear over the excitation range chosen, and only small differences between pitch and yaw axes were seen. Phase relations of various response points for three modes were also measured.

## INTRODUCTION

The design of the Atlas-Centaur launch vehicle has evolved as a relatively long, thin-walled pressurized cylinder (fig. 1). The control of this structure in flight results in the interaction of the vehicle lateral bending modes with the flight control system. Such coupling exists because the control system attitude sensors detect angular position and rates caused by structural deformations. In the case of the Atlas-Centaur, the lower-frequency bending modes are within the control system response range for a large portion of powered flight. The stability of the vehicle control, therefore, requires examination of the corresponding closed-loop modes of the vehicle structure and the control system. A schematic of the control system is shown at the top of the next page.



It was the purpose of the lateral dynamics tests to experimentally determine the frequencies, mode shapes, and damping characteristics of the Atlas-Centaur-Surveyor at various simulated flight times in order that accurate elastic modal data could be incorporated in the elastic stability analysis as well as in the analysis for loads and clearance. The complexity of the Atlas-Centaur structure, with its many discontinuities and branches, makes analytical predictions difficult. A test of the lateral bending dynamics of the vehicle was made to improve the analysis techniques and/or to gain confidence in these techniques.

## APPARATUS AND PROCEDURE

The tests were conducted on a full-scale, production-type Atlas-Centaur vehicle and a dynamic model of the Surveyor spacecraft. The vehicle was suspended on soft springs to simulate the free-free condition of flight and excited laterally at the gimbal plane with an electrodynamic shaker. Natural frequencies, mode shapes, and damping characteristics of the vehicle were studied. The mass distribution of the propellants was approximated with water in the Atlas liquid-oxygen and fuel tanks and the Centaur liquid-oxygen tank. Polystyrene balls were used to simulate the mass of the liquid-hydrogen fuel in Centaur.

### Test Setup

All testing described herein was accomplished at the E site of Plum Brook Station. The E site consists of a 140-foot- (42.6-m-) high steel tower designed to accommodate the

Atlas-Centaur vehicle (see fig. 2).

The test vehicle consisted of a complete Atlas-Centaur launch vehicle including nose fairing and Centaur insulation panels. A dynamic model of the Surveyor spacecraft was used in all tests. To facilitate mating with the suspension system, the Atlas was modified by replacing all structure and components aft of the thrust barrel with an equivalent mass beam structure X-frame (see fig. 2). The bending and shear stiffnesses of the vehicle are given in figures 3 and 4 and the mass distribution of the empty vehicle is given in figure 5.

A suspender consisting of a steel cable, a spring box, a hydraulic cylinder, and a load cell was fastened to each of the four frame ends (fig. 2). Each spring box contained 4 to 16 springs with a constant of about 400 pounds per inch ( $7.0 \times 10^4$  N/m) per spring. The number of springs for each test was proportioned to the vehicle weight for the particular configuration to give a static deflection of approximately 1 foot (0.3 m), keeping the natural rocking frequency of the suspension system an order of magnitude below the first bending mode frequency of the vehicle. Lateral stability of the system was provided at the bottom by the  $1/2^\circ$  ( $8.7 \times 10^{-3}$  rad) inclination of the cables and at the top by horizontal springs (fig. 2). The test setup was similar to that used in the longitudinal dynamics tests in reference 1.

To avoid operational problems involved with the handling of cryogenic propellants, an equal volume of deionized water was used instead of liquid oxygen and RP fuel. Polystyrene balls having an equivalent bulk density replaced the liquid hydrogen. With the exception of the first test, the propellant tanks were maintained at flight pressures of 31.0 and 59 psig ( $21.3 \times 10^4$  and  $40.6 \times 10^4$  N/m<sup>2</sup> gage) in the Atlas liquid-oxygen and fuel tanks, respectively, and 15 and 5 psig ( $10.3 \times 10^4$  and  $3.44 \times 10^4$  N/m<sup>2</sup> gage) in the Centaur liquid-oxygen and liquid-hydrogen tanks, respectively.

The tank pressures for the first test were maintained at the standby conditions of 17 psig ( $11.7 \times 10^4$  N/m<sup>2</sup>) in the Atlas fuel tank, 10 psig ( $6.89 \times 10^4$  N/m<sup>2</sup>) in the Atlas liquid-oxygen tank, 10 psig ( $6.89 \times 10^4$  N/m<sup>2</sup>) in the Centaur liquid-oxygen tank, and 5 psig ( $3.44 \times 10^4$  N/m<sup>2</sup>) in the Centaur fuel tank. The weight and levels of the simulated propellants are given in table I, along with tank pressures used during the tests.

## Instrumentation

Instrumentation for all tests consisted of strain-gage-type accelerometers. Locations on the X-axis and Y-axis sides are shown in figure 6. The accelerometers sensed acceleration perpendicular to the longitudinal (Z) axis of the vehicle. In the area of the Centaur insulation panels, holes were cut in the panels to allow attachment of the accelerometers to the tank skin. Load cells were used for both vehicle weighing and measurement of the driving force. All modal data were digitally recorded and reduced with a

digital computer program. Sixteen channels of data were recorded on oscillograph paper. All damping was determined by the logarithmic decay method from the analog traces on oscillograph paper. End-to-end system accuracy of the instruments is considered to be approximately 2 percent of full scale. Full-scale ranges of instruments were as follows:

Accelerometers, g (m/sec <sup>2</sup> ) . . . . .	±1.0 (±9.8)
Load cell (force), lb (N) . . . . .	±1000 (±4.45×10 <sup>3</sup> )

### Test Procedure

The vehicle tanks (except for the Centaur hydrogen tank) were filled with water and pressurized to simulate the flight conditions as presented in table I. Excitation was then applied to the suspended vehicle by an electrodynamic shaker through a load cell and the X-frame at frequencies varying from 1 to 40 hertz. Input force levels ranged from 30 to 400 pounds (133 to 1779 N).

Inasmuch as a tank rupture hazard existed at the tank pressures used, it was necessary to control all operations remotely once the tanking procedures were begun. These remote operations were conducted from a control room (H-building) located approximately 1/4 mile (400 m) away. Television cameras were used to visually monitor the vehicle.

The resonant frequencies of the vehicle were determined by slowly increasing the excitation frequency while holding the force input constant. Resonance was established when acceleration at the extremities of the vehicle peaked.

When resonant conditions were determined, transducer output was recorded on analog recorders and on digital tape. At the resonance peak, the shaker was electrically decoupled allowing natural decay of the oscillations, with the transducer output being recorded on the analog recorder. This procedure was followed to identify modes for each tanking condition.

### RESULTS AND DISCUSSIONS

The test results are discussed in terms of the following major subjects:

- (1) Natural frequencies
- (2) Mode shapes
- (3) Damping
- (4) Linearity of response
- (5) Phase relations of various parts of the mode

## Natural Frequencies

Figure 7 illustrates the variation of natural frequency of each mode with the simulated propellant level associated with vehicle flight times. The variation is shown only for the pitch (Y-Z) plane. Tables II lists natural frequencies as a function of flight time, plane of excitation, and pressure level. It is noted that although tank pods exist in the pitch plane and not in the yaw plane, the effect on natural frequency is minor. Further, two tests run at significantly different pressures show little variation in natural frequency. This is probably due to the linearity of this structure as is discussed later.

## Mode Shapes

Mode shapes measured at each of the natural frequencies are plotted in figures 8(a) to (i). The mode shapes are shown for both the pitch (Y-Z) and yaw (X-Z) planes and for two tank pressures for the empty vehicle configurations. Although mode shapes are easily defined as clean, continuous curves in the lower-frequency modes, the higher-frequency modal curves show breaks in the smoothness of the mode shape. A comparison of mode shapes in the two planes indicates that although lower modes compare reasonably, higher modes are significantly different in shape. The shapes of the first two modes in the pitch (Y-Z) plane at two significantly different tank pressures compare reasonably well, as demonstrated by figures 8(b) and (c).

## Damping

One of the most difficult parameters to measure in a test of this nature is damping. Even though every effort was made in the design and operation of the test setup to assure that no damping was introduced through the manner in which the vehicle was suspended, no quantitative measure of test-stand-introduced damping was possible. The typical decays shown in figure 9 present an example of the data from which damping was derived. Damping was determined by the logarithmic decay method (ref. 2). A plot of the damping variation with flight time in the Y-Z plane (pitch) is shown in figure 10. A typical value of damping in the pitch plane obtained from flight data during the booster engine cutoff (BECO) transient (T + 150 sec) is also plotted in figure 10. A damping value of 3 percent obtained from flight data compares to 2.2 percent measured in testing at this same simulated flight time.

## Linearity of Response

One of the more significant problems associated with analysis of control systems and loads interacting with the flexible-body bending modes is the linearity of these modes. Nonlinearities in modal response relative to forcing function level could dictate either convergence or divergence of loads or autopilot stability. Responses of the first four bending modes as a function of force level are shown in figure 11. The data shown in figure 11 indicate the modal response is linear for each of the modes over the excitation range chosen.

## Phase Relations of Various Parts of Mode

It is necessary to measure angular position and rate at a number of positions on a typical launch vehicle in order to provide the right amplitude and phase of bending mode relations in the autopilot. In the analysis of these control systems, orthogonal modes are assumed; that is, phase relations between any two parts of a vehicle are assumed to be either  $0^{\circ}$  or  $180^{\circ}$  for any vehicle mode, depending on the shape of the bending mode. To evaluate the limitations of this analysis technique, phase relations between force and acceleration were measured for frequencies near each of the first three bending modes. Figure 12 indicates the amount of phase differences associated with each of these frequencies. As the figure shows, very little deviation from orthogonality occurs in the first two modes. However, the third mode indicates significant differences in phase relations.

## CONCLUSIONS

Based on the results of the lateral dynamics tests of the Atlas-Centaur-Surveyor vehicle, the following conclusions are drawn:

1. The natural bending mode frequency is only slightly affected by tank pressure.
2. The bending mode frequency is essentially the same for excitation of the vehicle in the XZ and YZ planes.
3. The mode shapes for the first two modes are well defined, while the higher modes become less well defined.
4. Damping of the vehicle averaged about 2 percent of critical.
5. Modal response is indicated to be linear for each of the modes over the excitation range.

6. Very little deviation from orthogonality is indicated in the first two modes. The third and higher modes indicate significant deviation in phase relations.

Lewis Research Center,  
National Aeronautics and Space Administration,  
Cleveland, Ohio, April 8, 1969,  
491-05-00-03-22.

## REFERENCES

1. Gerus, Theodore F.; Housely, John A.; and Kusic, George: Atlas-Centaur-Surveyor Longitudinal Dynamics Tests. NASA TM X-1459, 1967.
2. Harris, Cyril M.; and Crede, Charles E.: Shock and Vibration Handbook. Vol. 1. McGraw-Hill Book Co., Inc., 1961.

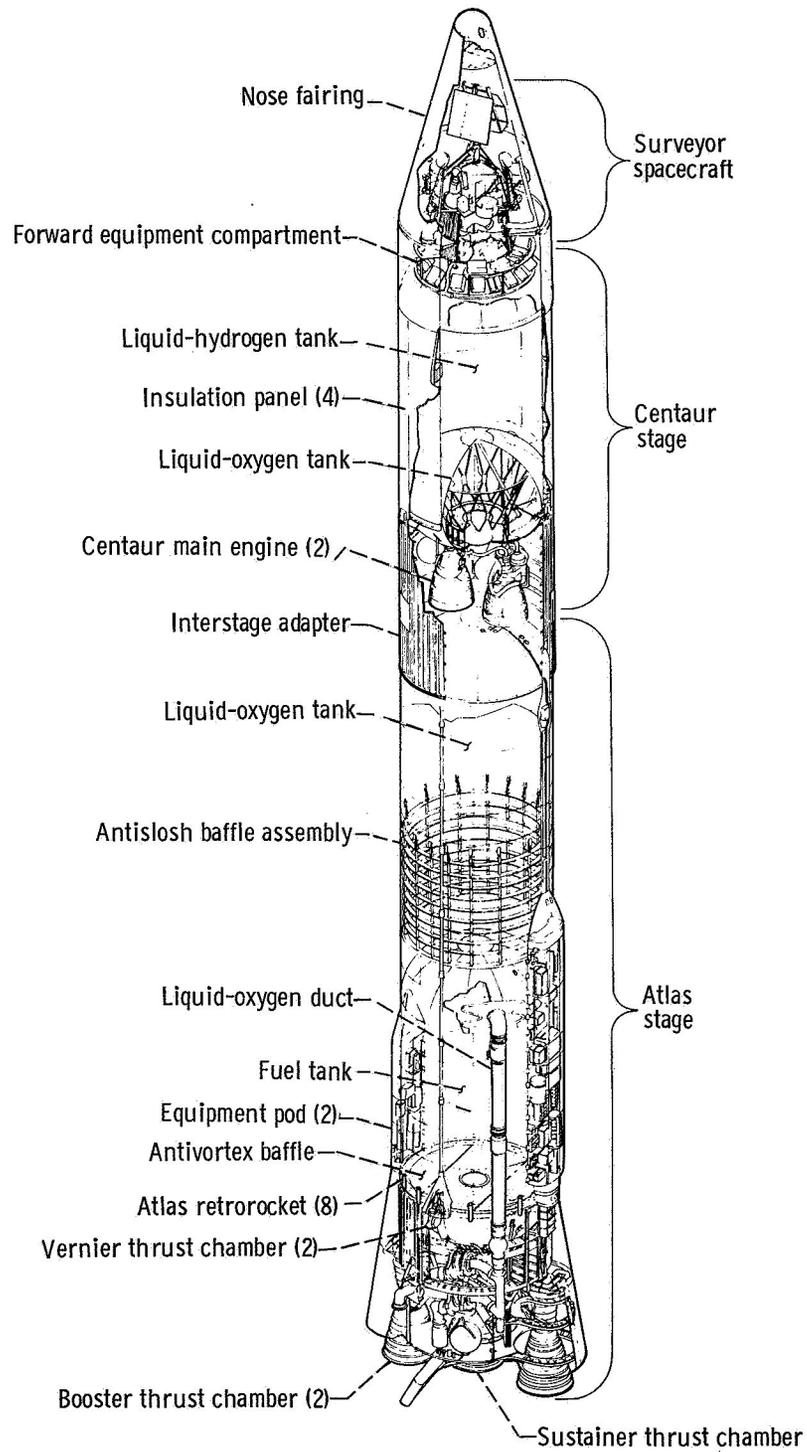
TABLE I. - TEST CONDITIONS

Test configuration	Units	Test number and simulated flight time					
		I	II and VII	III and VIII	IV	V and IX	VI and X
		Tanks empty	Tanks empty	T + 150 (max. g)	T + 100	T + 75 (max. $\alpha Q$ )	T + 0 (lift-off)
Atlas fuel tank							
Weight of water	lb	0	0	10 050	38 500	51 000	93 000
	N	0	0	44 600	171 000	226 000	412 000
Water level	Station	0	0	1144	1077	1077	940
Tank pressure	psig	17	59	59	59	59	59
	N/m <sup>2</sup>	11.68×10 <sup>4</sup>	40.5×10 <sup>4</sup>	40.5×10 <sup>4</sup>	40.5×10 <sup>4</sup>	40.5×10 <sup>4</sup>	40.5×10 <sup>4</sup>
Atlas liquid-oxygen tank							
Weight of water	lb	0	0	14 600	58 300	81 200	150 000
	N	0	0	64 700	258 200	360 000	665 000
Water level	Station	0	0	895	786	734	556
Tank pressure	psig	10.0	31.0	31.0	31.0	31.0	31.0
	N/m <sup>2</sup>	6.86×10 <sup>4</sup>	21.25×10 <sup>4</sup>	21.25×10 <sup>4</sup>	21.25×10 <sup>4</sup>	21.25×10 <sup>4</sup>	21.25×10 <sup>4</sup>
Centaur liquid-oxygen tank <sup>a</sup>							
Weight of water	lb	0	0	21 400	21 400	21 400	21 400
	N	0	0	95 000	95 000	95 000	95 000
Water level	Station	0	0	382	382	382	382
Tank pressure	psig	10.3	15.0	15.0	15.0	15.0	15.0
	N/m <sup>2</sup>	6.86×10 <sup>4</sup>	10.3×10 <sup>4</sup>	10.3×10 <sup>4</sup>	10.3×10 <sup>4</sup>	10.3×10 <sup>4</sup>	10.3×10 <sup>4</sup>
Total weight (including weight of tanks)	lb	25 530	25 530	71 580	143 730	179 130	289 930
	N	113 200	113 200	318 000	636 000	795 000	1 285 000

<sup>a</sup>Conditions of Centaur liquid-hydrogen tank were the same for all tests: weight of polystyrene, 4630 pounds (20 500 N); level of polystyrene, station 209; tank pressure, 5.0 psig (3.43×10<sup>4</sup> N/m<sup>2</sup>).

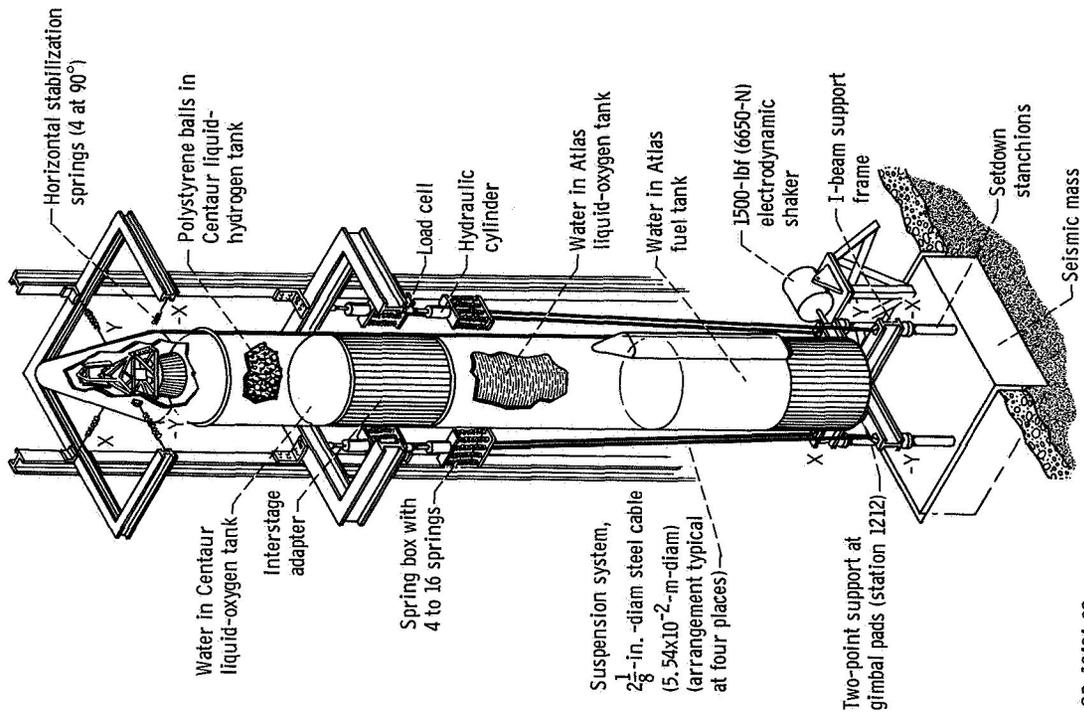
TABLE II. - EXPERIMENTAL NATURAL FREQUENCIES AND DAMPING RATIOS FOR TEST SIMULATED FLIGHT CONDITIONS

(a) Excitation force in Y-Z plane				(b) Excitation force in X-Z plane			
Test condition	Mode	Frequency, Hz	Damping ratio	Test condition	Mode	Frequency, Hz	Damping ratio
Oxidizer and fuel tanks empty and at standby pressure	1	6.21	0.022	Oxidizer and fuel tanks empty at flight pressures	1	6.21	0.037
	2	14.20	.024		2	13.49	.058
Oxidizer and fuel tanks empty at nominal flight pressures	1	6.12	0.023		3	31.40	No data
	2	14.51	.053	Simulated flight conditions at T + 150 seconds	1	4.74	0.035
	3	31.90	.023		2	10.78	No data
	4	37.00	.022		3	26.62	0.025
Simulated flight conditions at T + 150 seconds	1	4.93	0.022		4	36.60	No data
	2	9.89	.033	Simulated flight conditions at T + 75 seconds	1	2.97	0.026
	3	25.71	.014		2	8.01	No data
	4	35.09	No data		3	13.35	0.014
Simulated flight conditions at T + 100 seconds	1	3.58	0.025		4	20.60	.006
	2	8.13	.011		5	32.89	.016
	3	17.42	.012	Simulated conditions at T + 0 (lift-off)	1	2.60	0.023
	4	23.12	.009		2	6.52	.021
Simulated flight conditions at T + 75 seconds	1	3.00	0.024		3	10.01	.022
	2	7.77	.013		4	13.01	.026
	3	13.44	.012		5	17.90	.006
	4	20.37	.031	Simulated conditions at T + 0 (lift-off)	1	2.60	0.023
	5	31.44	.002		2	6.52	.021
Simulated conditions at T + 0 (lift-off)	1	2.59	0.019		3	10.01	.022
	2	6.49	.013		4	13.01	.026
	3	9.82	.019		5	17.90	.006
	4	12.74	No data				
	5	17.98	0.004				



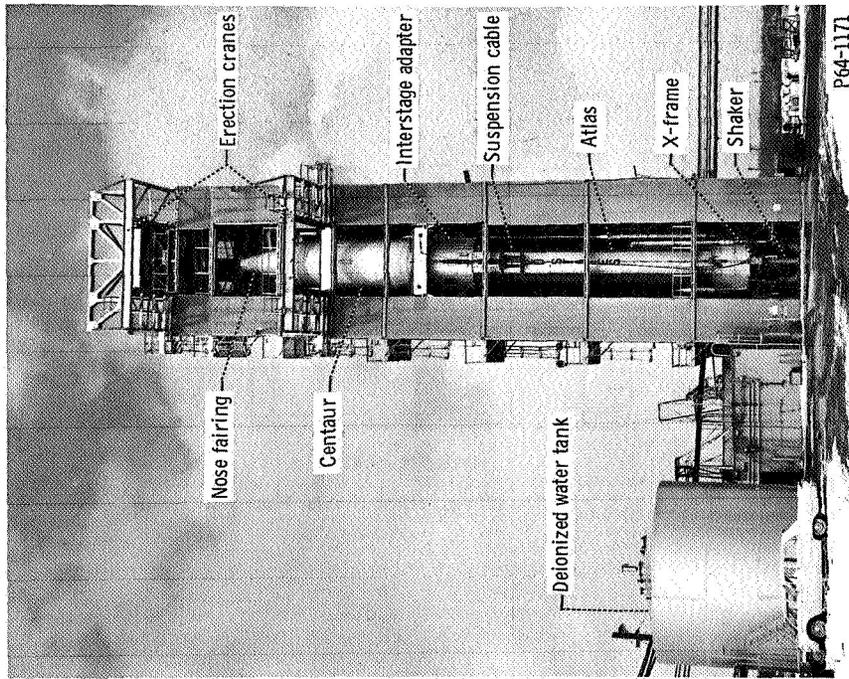
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Figure 1. - Atlas-Centaur-Surveyor space vehicle configuration.



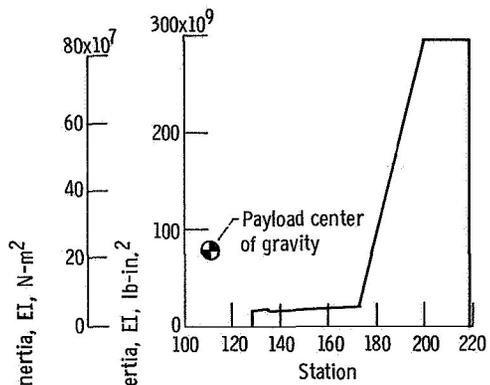
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(b) Vehicle support system.

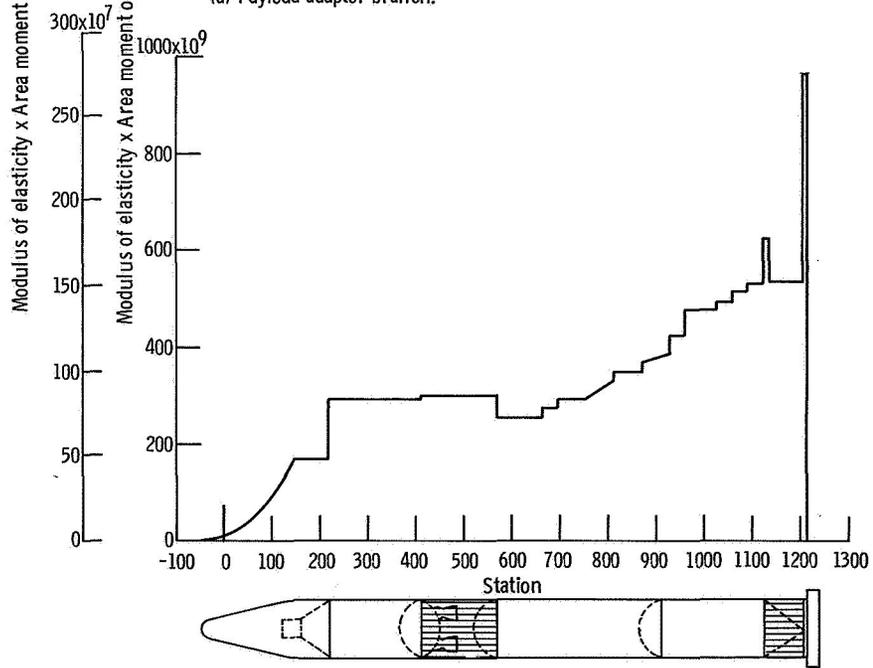


(a) E-stand test facility with Atlas-Centaur-Surveyor.

Figure 2. - Test setup.

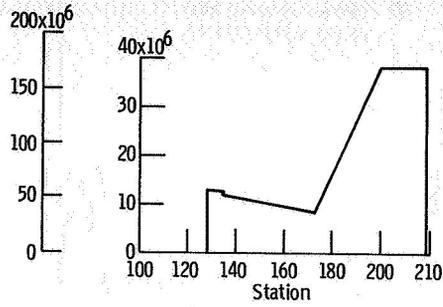


(a) Payload adapter branch.

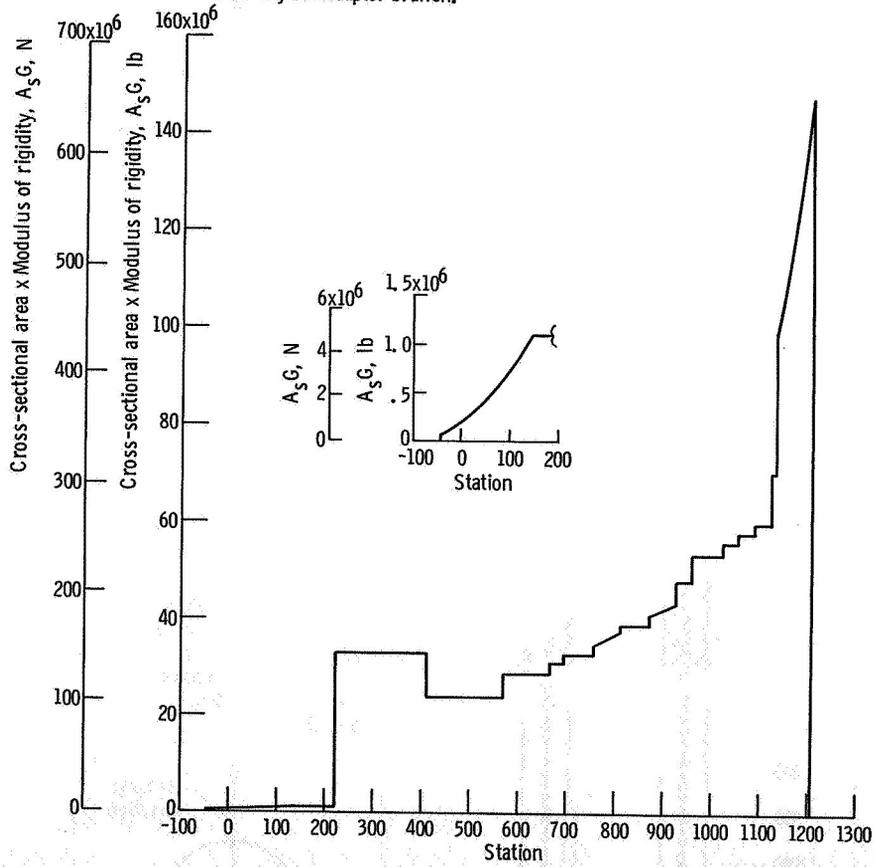


(b) Entire vehicle.

Figure 3. - Bending stiffness distribution for Plum Brook vehicle.



(a) Payload adapter branch.



(b) Entire vehicle.

Figure 4. - Shearing stiffness distribution for Plum Brook vehicle.

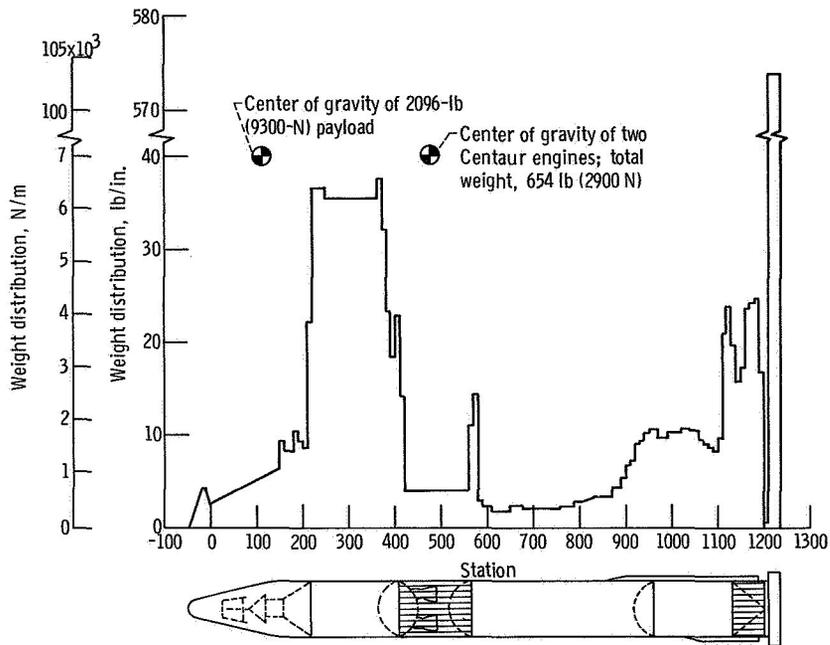


Figure 5. - Weight distribution as function of station for test I configuration.

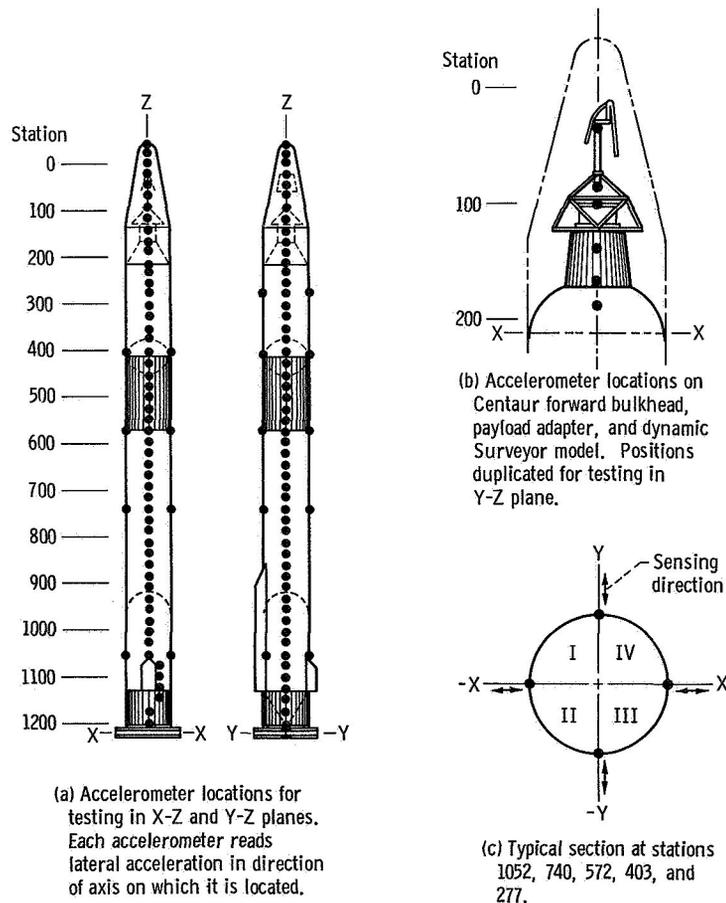


Figure 6. - Location of accelerometers.

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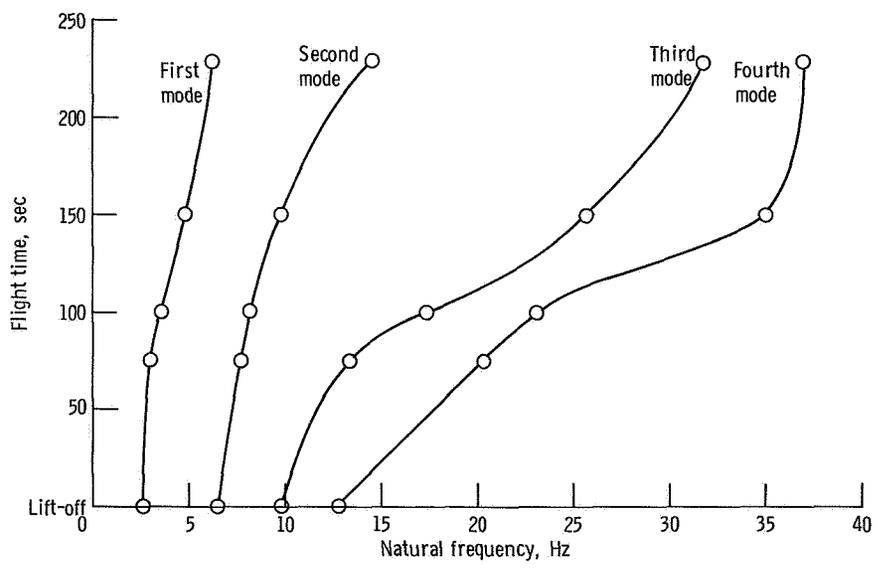
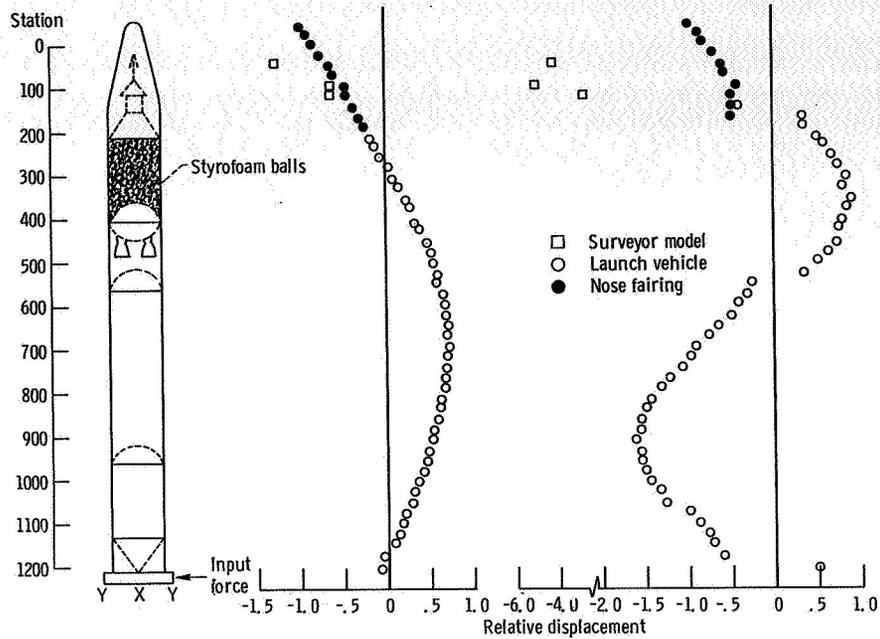


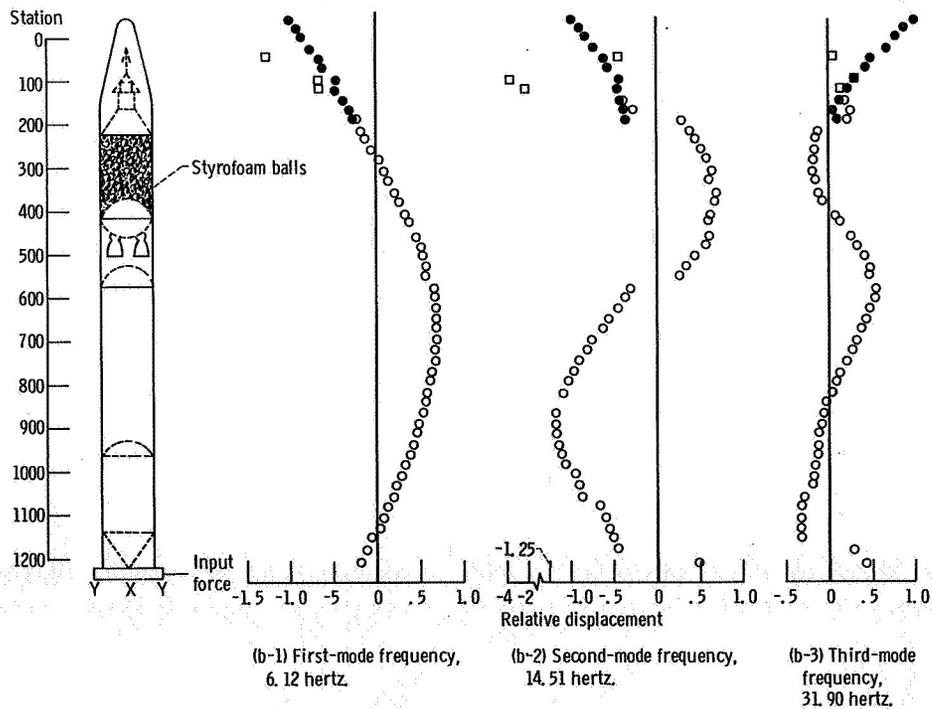
Figure 7. - Variation of natural frequency as function of simulated flight time.



(a-1) First-mode frequency, 5.95 hertz.

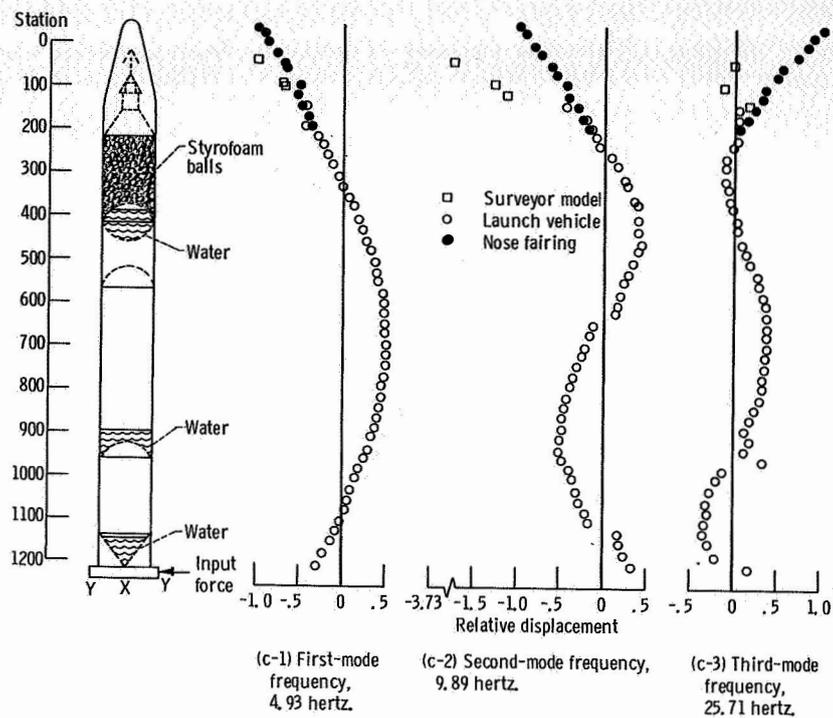
(a-2) Second-mode frequency, 14.17 hertz.

(a) Oxidizer and fuel tanks empty at standby pressures (10 and 17 psig ( $6.89 \times 10^4$  and  $11.7 \times 10^4$  N/m<sup>2</sup> gage), respectively); Y-Z plane.

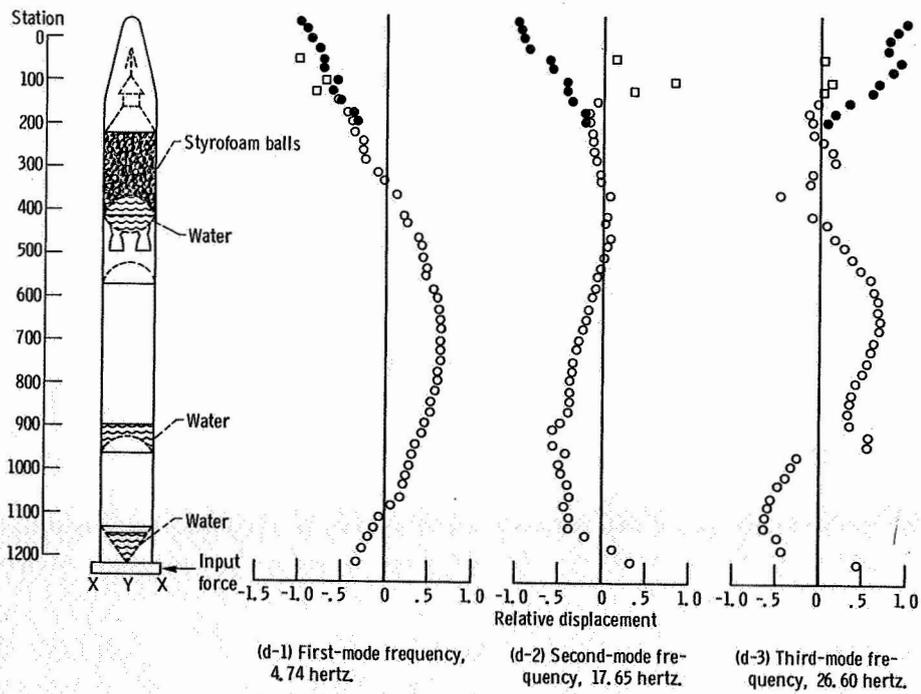


(b) Oxidizer and fuel tanks empty at flight pressure; Y-Z plane.

Figure 8. - Lateral dynamics - bending mode shapes. Surveyor dynamic model SD-4.

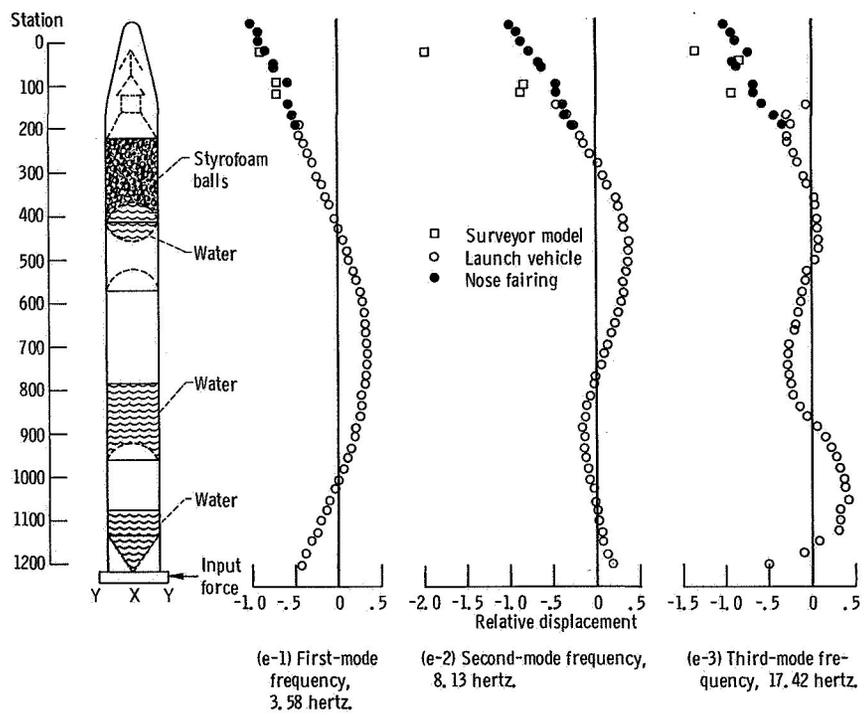


(c) Simulated flight conditions at T + 150 seconds; Y-Z plane.



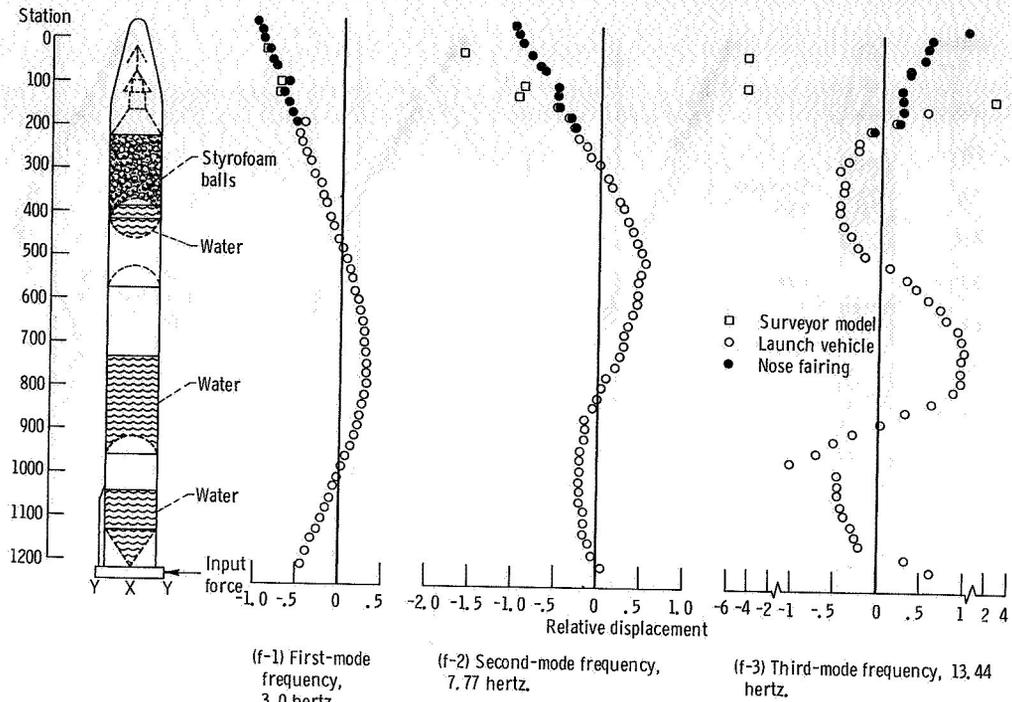
(d) Simulated flight conditions at T + 150 seconds; X-Z plane.

Figure 8. - Continued.

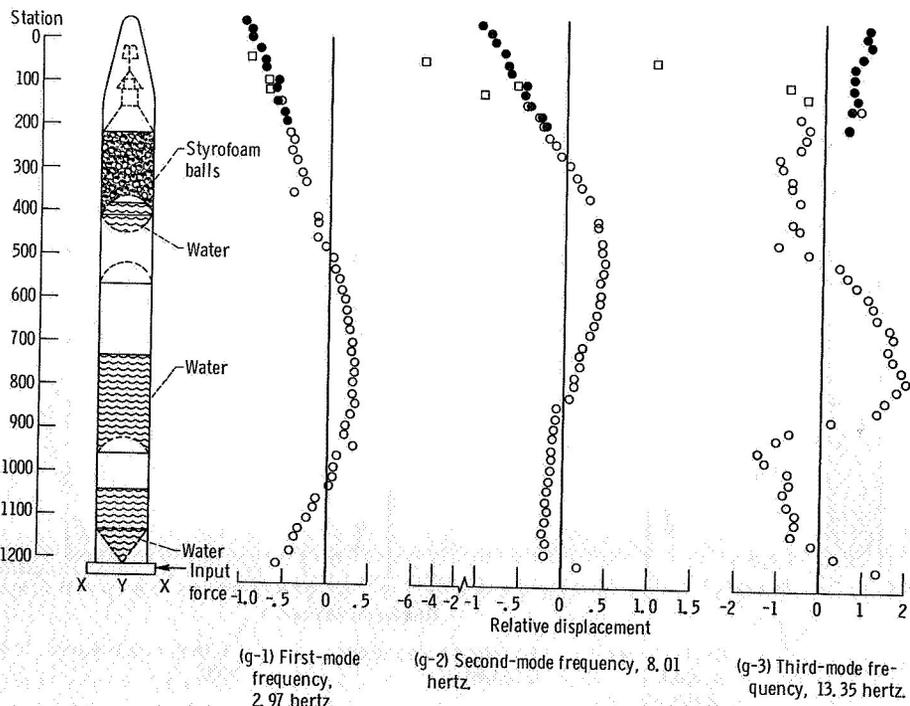


(e) Simulated flight conditions at T + 100 seconds; Y-Z plane.

Figure 8. - Continued.



(f) Simulated flight conditions at T + 75 seconds; Y-Z plane.



(g) Simulated flight conditions at T + 75 seconds; X-Z plane.

Figure 8. - Continued.

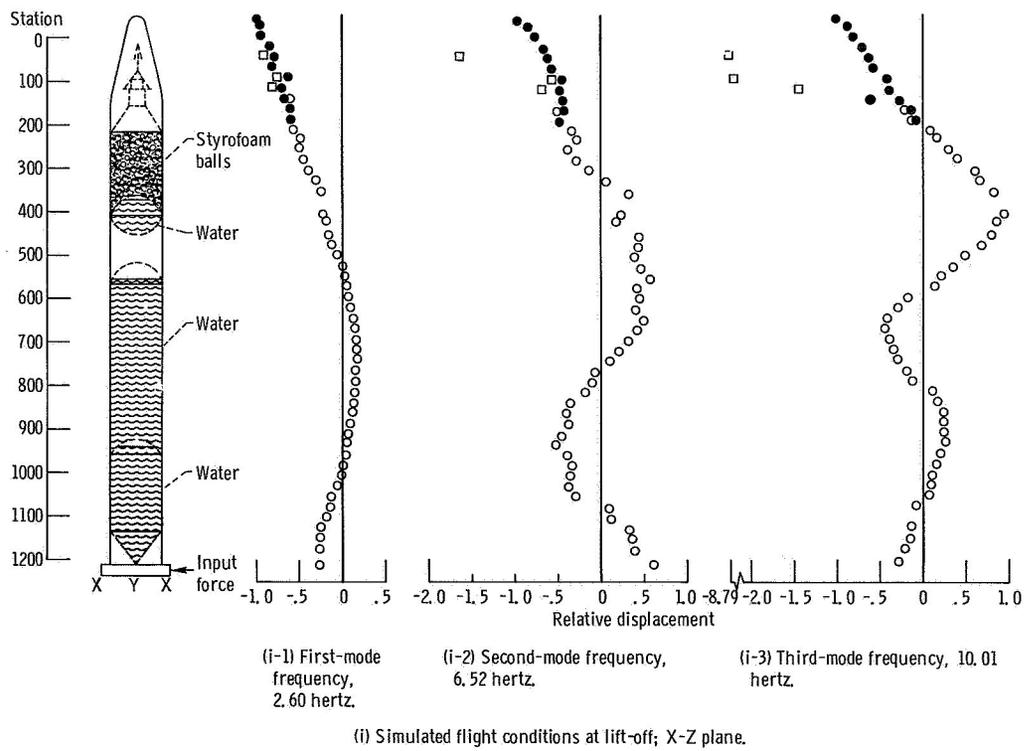
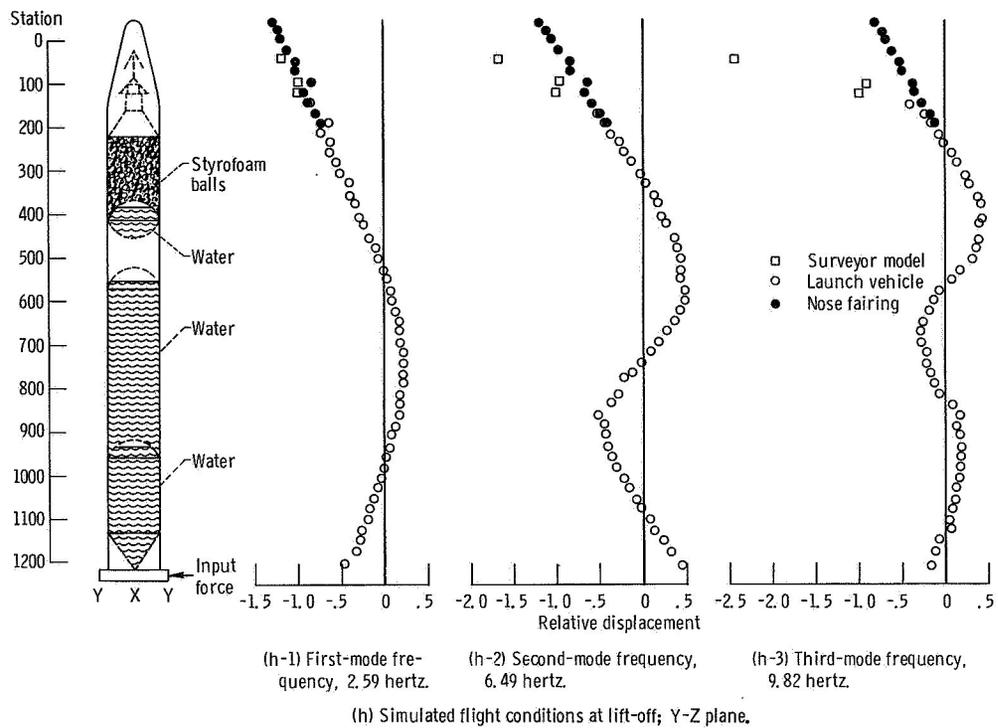


Figure 8. - Concluded.

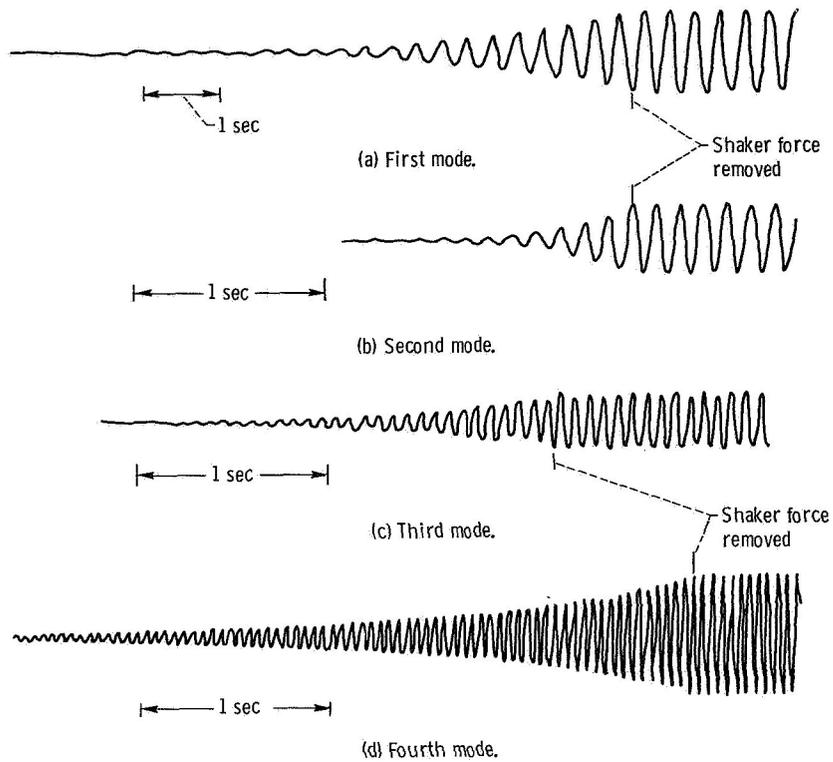


Figure 9. - Typical data used to calculate damping by decay methods in Atlas-Centaur-Surveyor lateral dynamics tests. Simulated flight conditions at T + 75 seconds.

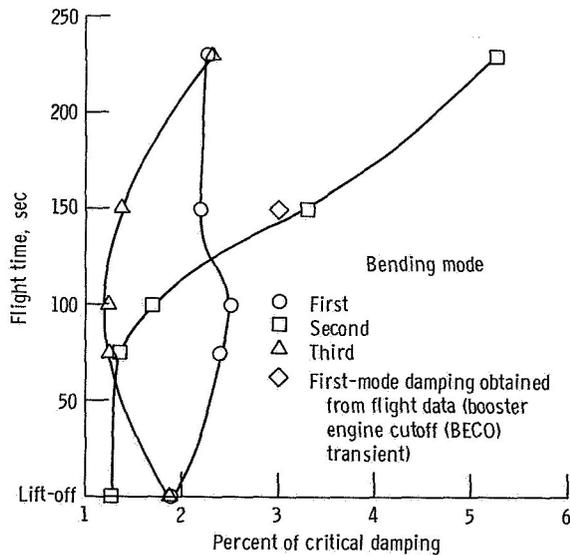


Figure 10. - Percent of critical damping as function of flight time; Y-Z plane (pitch).

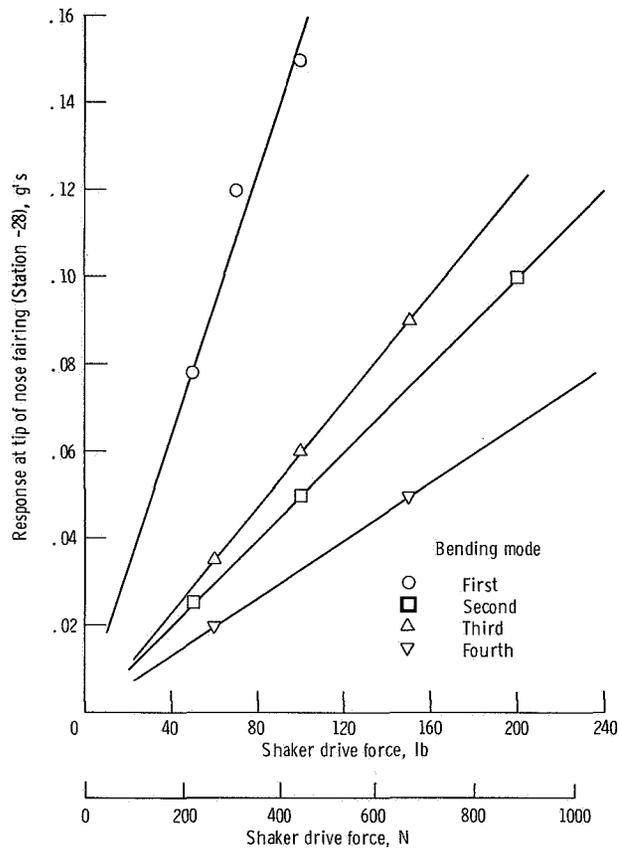


Figure 11. - Response at tip of nose fairing as function of shaker drive force. Simulated flight conditions at T + 75 seconds.

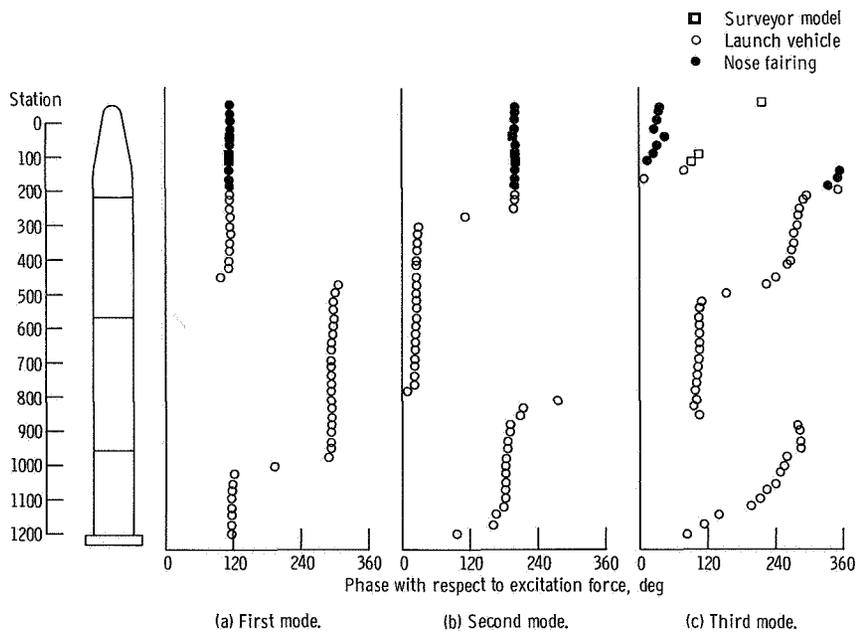


Figure 12. - Phase with respect to excitation force as function of vehicle station. Simulated flight conditions at T + 75 seconds; Y-Z plane.

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