TO: KSI/Scientific & Technical Information Division
Attn: Miss Winnie M. Morgan

FROM: GP/Office of Assistant General Counsel for Patent Matters

SUBJECT: Announcement of NASA-Owned U.S. Patents in STAR

In accordance with the procedures agreed upon by Code GP and Code KSI, the attached NASA-owned U.S. Patent is being forwarded for abstracting and announcement in NASA STAR.

The following information is provided:

U.S. Patent No. : 3,678,771

Government or Corporate Employee : U.S. Government

Supplementary Corporate Source (if applicable) :

NASA Patent Case No. : MFS-70,645-1

NOTE - If this patent covers an invention made by a corporate employee of a NASA Contractor, the following is applicable:

YES ☑  NO ☒

Pursuant to Section 305(a) of the National Aeronautics and Space Act, the name of the Administrator of NASA appears on the first page of the patent; however, the name of the actual inventor (author) appears at the heading of column No. 1 of the Specification, following the words "...with respect to an invention of ..."

Bonnie L. Woerner

Enclosure

NOTE: LATE ISSUE PICK-UP. Was never published as patent.
A two-speed drive system for driving an independently driven vehicle wheel comprising a relatively large wheel sprocket attached to the vehicle wheel. A drive belt passes around the large wheel sprocket and one of a pair of smaller, selectively interchangeable, motor-driven wheels. The two smaller wheels are of the same diameter but rotate at different speeds and are mounted on a plate that may be temporarily released through a latching pawl for rotation of 180° so that the two wheels change places. No external force is required to rotate the plate, when released, since one of the smaller wheels turning in contact with the drive belt causes the plate to rotate rather than the drive belt and large wheel sprocket. When the drive belt is engaging both of the smaller wheels simultaneously during the shifting phase, the normally slower turning wheel is permitted to turn faster through a ratchet mechanism.
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BRIEF DESCRIPTION OF THE DRAWING

FIG. 1 is a pictorial view showing a two-speed drive system arranged in the low-speed position and installed on the inside of a spoke wheel for driving the wheel.

FIG. 2 is a schematic elevational view of the two-speed drive system of FIG. 1 taken generally along line 2-2 of FIG. 1 and showing the drive system in the low-speed position.

FIG. 3 is a view similar to FIG. 2 but showing the drive system during the shifting phase from one speed to the other.

FIG. 4 is a view similar to FIG. 2 but showing the drive system in the high-speed position.

FIG. 5 is an enlarged view, partially in section, showing a ratchet mechanism operative during the speed shifting phase for permitting the low-speed power transmitting wheel to turn at a faster than normal speed.

DESCRIPTION OF THE PREFERRED EMBODIMENT

Referring to FIG. 1, there is shown a wheel drive sprocket 11 attached to spokes 13 of a driven wheel 14 which may be one wheel of a vehicle wherein the wheels of the vehicle are individually driven through the two-speed drive system of the present invention. The spokes 13 radially radiate from a wheel hub 15 that turns on an axle 17 and the spokes are attached to the sprocket 11 by conventional clamps (not visible in the drawing).

Welded to the axle 17 is a triangular bracket plate 19 reinforced by a corner brace 21. The bracket 19 has a pair of extensions 23, each having an end of a fixed shaft 25 joined thereto and on which an idler 27 turns. Each of the shafts 25 is fixed at the other end to a relatively large stationary plate member 29. A rotatable circular plate member 33 is pivotally joined at its center to the stationary plate member 29 by a pivot pin 35. A portion of the upper edge of the stationary plate member 29 is circular so that it corresponds to and is flush with the circular plate member 33. Aligned notches 37 and 39 are provided in the plates 29 and 33, respectively, and a spring-latched retaining pawl 43 reciprocates within the notches 37 and 39 to selectively release the plate member 33 for rotation around the pivot pin 35. Two of the notches 39 are formed in the plate member 33 180° apart. The latching pawl 43 is mounted on the plate member 29 by a bracket 45 and is actuated through a cable 47 extending to the desired operating location.

A drive shaft 51 has one end thereof rotatably mounted in the plate member 33, and fixed to the drive shaft 51 is a power transmitting wheel 53. The drive shaft 51 extends perpendicular to the plate 33 to an electric motor 55 that selectively rotates the shaft 51. The motor 55 is pivotally joined to a bracket 57 through a fitting 59 so that the motor 55 may rotate eccentrically counterclockwise around an axis that coincides with a line defining the rotational axis of the plate 33 as will be explained hereinafter.

A second shaft 63 is rotatably mounted at one end in the plate 33 and at the other end to the motor 55, extending parallel with the drive shaft 51. The shaft 63 is merely rotatably mounted to the housing of the motor 55 and is not directly driven by the motor 55. Power transmitting wheel 65 equal in diameter and length to the wheel 53 is fixed to the shaft 63. The wheel 65 over a portion thereof is joined to a step-down wheel 69 of larger diameter than the wheels 53 and 65. The pair of wheels 53 and 65 and the shafts 51 and 63 are diametrically aligned with each other and with the pivot pin 35, with the pivot pin 35 being located midway between the shafts 51 and 63. The pair of wheels 53 and 65 are also in a common plane that extends perpendicular to the shafts 51 and 63.

A ratchet drive is incorporated between the wheel 65 and the step-down wheel 69. As shown in FIG. 5, the ratchet mechanism comprises a series of teeth 71 formed around the inner surface of the step-down wheel 69 and a plurality of pawls 73 projecting from the wheel 65 and engaging the teeth 71. As is apparent, the wheel 65 may turn clockwise within the step-down sprocket 69, the reason for which will become ap-
As shown in FIG. 1, a drive belt 75 passes around the wheel sprocket 11 and the wheel 65. The other side of the belt 75 passes over the idlers 27. A belt 77 passes around the step-down wheel 69 and a portion of the wheel 53. The remaining portion of the wheel 53 is not in contact with either belt in the position shown in FIG. 1.

It is evident that the electric motor 55 may be positioned very close to the wheels 53 and 61, making the shafts 51 and 63 very short. In FIG. 1 the motor 55 is shown spaced from the wheels 53 and 61 primarily for purposes of clarity in the illustration.

**OPERATION**

As shown in FIGS. 1 and 2, the drive system is in the low speed position and the belts 75 and 77 would be moving in the clockwise direction as indicated by the arrows. The drive shaft 31 of the motor 55 turns the wheel 53 clockwise which drives the step-down wheel 69 through the belt 77. The step-down wheel 69, being larger in diameter than the wheel 53, turns at a lower rpm than the wheel 53. The wheel 65 turns at the same rpm as the step-down wheel 69 since it is connected to the wheel 69 through the ratchet mechanism shown in FIG. 5. Thus in the position of FIGS. 1 and 2 the wheel 65 drives the wheel sprocket 11 through the drive belt 75 at a relatively low speed.

When it is desired to shift the drive system to the relatively high-speed position illustrated in FIG. 4, the spring-urged latching pawl 43 is retracted by the cable 47 to unlock the rotatable plate 33 from the plate 29. Since the plate 33 is then free to rotate, the wheel 65, turning in contact with the drive belt 75, will cause the plate 33 to rotate counterclockwise rather than forcing the drive belt 75 to rotate clockwise. Thus the wheel 65 rolls along the inner surface of the drive belt. As the plate 33 rotates counterclockwise the drive belt 75 will contact simultaneously both of the wheels 53 and 65, as indicated in FIG. 3, which shows the drive system in the transitory position between the low-speed and high-speed positions.

When the drive belt 75 is in contact simultaneously both of the wheels 53 and 65 it is necessary for both of these wheels to turn at the same speed. The ratchet mechanism between the wheel 65 and the step-down wheel 69 shown in FIG. 5 permits the wheel 65 to turn clockwise within the step-down wheel 69 as the pawls 73 ride over the teeth 71. Therefore, the wheel 65 may turn at the same speed as the wheel 53 during the time when both wheels are in contact with the drive belt 75 as shown in FIG. 3.

When the drive assembly is in the transitory position of FIG. 3 the plate 33 has rotated substantially 90° counterclockwise and will continue to rotate counterclockwise until it has rotated 180° at which time it will be automatically locked to the plate 29 by the spring-urged pawl 43 sliding into the notch 39. As previously indicated, the notches 39 are located 180° apart on the rotatable plate 33. After the full 180° rotation of the plate 33 the drive system will be locked in the high-speed position shown in FIG. 4. In this position the drive belt 75 passes over the wheel 53 that turns at a high speed relative to the wheel 65 over which the drive belt passed in the low-speed position of FIGS. 1 and 2. The wheel 65 is not in contact with the drive belt 75 in the high-speed position of FIG. 4 although it is still turning at a relatively low speed.

To change the drive system back to the low-speed position of FIGS. 1 and 2 it is only necessary to retract the latching pawl 43 and the plate 33 will rotate another 180° causing the pair of wheels 53 and 65 to exchange positions again. As the wheels 53 and 65 exchange places to shift from one speed to another the motor 55, supported by bracket 57, rotates eccentrically counterclockwise around an axial line passing through the pivot pin 35 just as do the wheels 53 and 65.

In the high-speed position of FIG. 4 the drive system may be driven in the reverse direction by reversing the motor 55. However, in the low-speed position of FIGS. 1 and 2 the drive system cannot be driven in reverse because the step-down wheel 69 would thus be turning counterclockwise and the ratchet teeth 71 (FIG. 5) would slide over the paws 73 without transferring the required torque to the wheel 65.

1. A two-speed drive system comprising:
   a. a rotatable member pivotally joined to a supporting member for rotation around a predetermined axis;
   b. means for locking said rotatable member against rotation around said predetermined axis;
   c. a pair of power transmitting wheels mounted on said rotatable member, said wheels being arranged for rotation around axes parallel with said predetermined axis;
   d. portions of said pair of wheels being in the same plane,
      said plane being perpendicular to said parallel axes;
   e. means for driving said pair of wheels, said means including means for driving one of said wheels at a lower rpm than the other of said wheels;
   f. a third rotatable wheel disposed coplanar with said portions of said pair of wheels and adapted to be driven selectively by either of said pair of wheels;
   g. a drive belt engaging said third wheel and one of said pair of wheels;
   h. the other of said pair of wheels being disengaged from said drive belt;
   i. means for selectively releasing said rotatable member for rotation around said predetermined axis whereby said rotatable member will rotate as said wheel engaging said drive belt turns in contact with said belt.

2. The invention as defined in claim 1 wherein said pair of wheels are of substantially equal diameter, said third wheel being larger in diameter than said pair of wheels, the said wheel being disengaged from said drive belt being disposed between said third wheel and said one of said pair of wheels engaging said drive belt.

3. The invention as defined in claim 2 wherein said predetermined axis of said rotatable member is located midway between and in diametrical alignment with the axes of rotation of said pair of wheels whereby said pair of wheels exchange position upon 180° rotation of said rotatable member, said locking means including means for automatically locking said rotatable member after rotation of 180°.

4. The invention as defined in claim 2 wherein said means for driving one of said pair of wheels at a lower rpm than the other of said pair of wheels comprises a step-down wheel joined coaxially with one of said pair of wheels, a drive belt engaging said step-down wheel and the other of said pair of said wheels, said step-down wheel being of larger diameter than either of said pair of wheels.

5. The invention as defined in claim 4 wherein said step-down wheel is joined to said one of said pair of wheels by means including means for permitting said one wheel to rotate at a different rpm than said step-down wheel.

6. The invention as defined in claim 2 wherein said means for driving said pair of wheels includes an electric motor, means for interconnecting said electric motor with one of said pair of wheels whereby power is transmitted from said motor to said one of said pair of wheels.

7. The invention as defined in claim 6 including means for supporting said electric motor for rotation around an axis in alignment with said predetermined axis.

8. The invention as defined in claim 2 wherein said rotatable member comprises a circular plate, said predetermined axis being located at the center of said plate, said means for locking said plate against rotation comprising two diametrically opposed notches in said plate.