DYNAMICS OF A GRAVITY-GRADIENT
STABILIZED FLEXIBLE SPACECRAFT

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NATIONAL AERONAUTICS AND SPACE ADMINISTRATION • WASHINGTON, D. C. • AUGUST 1974
This investigation is concerned with the dynamics of a gravity-gradient stabilized flexible satellite in the neighborhood of a deformed equilibrium configuration. First the equilibrium configuration is determined by solving a set of nonlinear differential equations. Then stability of motion about the deformed equilibrium is tested by means of the Liapunov direct method and the natural frequencies of oscillation of the complete structure calculated. The analysis is applicable to the RAE/B satellite.
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1. Introduction

With the advent of large spacecraft, flexibility has become an increasingly important factor in the system attitude stability. Early designs of spacecraft were based on rigid body analysis, according to which rotational motion is stable if it takes place about the axis of maximum or minimum moment of inertia and unstable if the body rotates about the axis of intermediate moment of inertia (see, for example, Ref. 1, Sec. 6.7).

The erratic behavior of the Explorer I, a satellite stabilized about the axis of minimum moment of inertia, prompted a re-examination of the rigid body assumption. Indeed, Thomson and Reiter\(^2\) were able to attribute the behavior of the Explorer I satellite to energy dissipation resulting from the vibration of flexible antennas. This conclusion was corroborated by Meirovitch.\(^3\) References 2 and 3 used the so-called "energy sink" approach. Their main conclusion was that a flexible satellite cannot be stabilized about the axis of minimum moment of inertia, leaving as stability criterion what has come to be known as the "greatest moment of inertia" requirement.

For a number of years, no significant additional work on the stability of flexible spacecraft was performed. Some work on cable-connected space stations cannot be really considered pertinent. Some investigation that can be regarded as being related to flexible spacecraft is that by Hooker and Margulies,\(^4\) who model a satellite as "a set of n rigid bodies interconnected by dissipative elastic joints," and forming so-called "topological trees."

The first serious attempt to treat rigorously the flexibility effects on the attitude stability of flexible satellites can be attributed to Meirovitch and Nelson.\(^5\) Reference 5 investigated a satellite with elastic appendages by means of an infinitesimal analysis. It appears that Ref. 5 uses modal analysis for the first time in conjunction with the
stability of flexible spacecraft. At the same time, Nelson and Meirovitch used the Liapunov direct method to investigate the stability of a rigid satellite with elastically connected moving parts. The motion of a satellite consisting of two rigid bodies connected by an elastic structure was investigated by Robe and Kane by means of an infinitesimal analysis. Simulating a spacecraft by a set of rigid masses interconnected by massless elastic members, Likins derived the corresponding equations of motion, and indicated that a solution can be obtained by modal analysis. Reference 8, however, does not produce an algorithm for the solution of the equations. Thermal effects and solar radiation pressure were found by Etkin and Hughes to cause the anomalous behavior of spinning satellites with long flexible antennas. The flexibility effects on the attitude motion of spacecraft were also investigated by Modi and Berenton but the validity of their analysis is in doubt, as they restrict the satellite vibration to planar.

An interesting paper by Newton and Farrell presents a method for evaluating the natural frequencies of a flexible gravity-gradient stabilized satellite. In the process, Reference 11 linearizes the equations of motion about the deformed equilibrium. As generalized coordinates, the investigators consider complete deformation patterns of the satellite. This procedure is not only unorthodox but also tends to limit the number of degrees of freedom of the simulation, not to mention the fact that one must guess in advance configuration patterns. Moreover, there is some question as to the evaluation of the equilibrium configuration. Nevertheless, the paper contains some interesting ideas. A paper by Likins and Wirsching proposes to introduce the concept of "synthetic modes" in conjunction with a "hybrid" coordinate system, where the latter is defined as a set of coordinates consisting of rotational coordinates of the spacecraft as a whole.
and modal coordinates for the flexible appendages. This idea, however, was introduced earlier in Reference 5.

All preceding investigations have one thing in common, namely, they are all discretization schemes. Some use lumping of the distributed mass of the elastic members, a procedure referred to sometimes as spatial discretization, and others use series truncation in conjunction with modal analysis. In a first attempt to apply Liapunov's direct method to hybrid systems from the area of satellite dynamics, i.e., without using any discretization scheme, Meirovitch studied the stability of spinning rigid bodies with elastic appendages. It should be pointed out that the term "hybrid" refers here to a system defined by sets of both ordinary and partial differential equations, a concept different from that used by Likins and Wirsching. Several new ideas were introduced in Ref. 13, such as the use of the bounding properties of Rayleigh's quotient to eliminate spatial derivatives from the problem formulation and the use of testing density functions.

The ideas of Refs. 13-15 have been pursued by Meirovitch and Calico for the case in which testing density functions cannot be defined readily. References 16 and 17 develop the so-called "method of integral coordinates," whereby certain integrals are identified and defined as generalized coordinates. Then, using the bounding properties of Rayleigh's quotient as well as certain Schwarz's inequalities for functions, a function \( \kappa \) bounding the Hamiltonian \( H \) from below is obtained, \( \kappa \leq H \), so that \( \kappa \) can be used as a testing function in conjunction with Liapunov's direct method. The method of integral coordinates is basically a discretization scheme.

One problem that has received little attention in the technical literature is that of deformed equilibrium, which can be referred to mathematically as "nontrivial equilibrium." Such problems arise in the case of gravity-gradient or spin-stabilized satellites with very flexible
appendages that are not aligned with the satellite's principal axes. Finding the equilibrium configuration can be quite a problem in itself, especially if the governing equations are nonlinear. Addressing himself to this problem, Flatley \(^{18}\) obtained the nonlinear equilibrium configuration of the Radio Astronomy Explorer (RAE) satellite by means of an analogue computer. Deformed equilibrium has also been considered in Ref. 11, but the details are not clear and no plot of the deformed equilibrium is shown. In seeking stability statements for the RAE/B satellite, Meirovitch \(^{19}\) obtained as a by-product the nonlinear deformed equilibrium, thus confirming the results of Ref. 18.

The present study is concerned with the stability of a hybrid dynamical system about nontrivial equilibrium. It contains many of the formulations and results of Ref. 19. Qualitative stability statements are obtained for the RAE/B satellite by both the Liapunov direct method and by an infinitesimal analysis. In connection with the infinitesimal analysis, the natural frequencies of oscillation about the nonlinear nontrivial equilibrium were obtained by a method developed by the first author of this report.\(^ {20}\) The method of Ref. 20 considers a state vector consisting of generalized coordinates and velocities, where the coordinates include both rotations and elastic deformations, and develops an eigenvalue problem in terms of real quantities alone. The stability statements of Ref. 19 and corresponding statements obtained from the solution of the eigenvalue problem agree completely.

2. **Problem Formulation**

We shall be concerned with the motion of a body consisting of \(n + 1\) parts, of which one part is rigid and \(n\) parts are elastic. The domain of
extension of the rigid part is denoted by $D_0$ and those of the elastic parts when in undeformed state by $D_i$ $(i = 1, 2, \ldots, n)$ (see Fig. 1). Correspondingly, the masses associated with the domains $D_i$ are denoted by $m_i$ $(i = 0, 1, \ldots, n)$, so that the total mass is $m = \sum_{i=0}^{n} m_i$. The elastic domains are rigidly attached to $D_0$ and have common boundaries only with $D_0$.

The body $m$ is assumed to move in a central-force gravitational field, with its mass center describing a given orbit about the center of force $C.F.$, where the latter is assumed to be fixed in an inertial space.

In describing the motion of $m$ it will prove convenient to identify a system of axes $xyz$ (see Fig. 1) with the undeformed state. The origin $c$ of $xyz$ is taken to coincide with the mass center of $m$ in the undeformed state and axes $xyz$ themselves coincide with the principal axes of $m$ in the same state. Note that the system $xyz$ is embedded in the rigid part $D_0$ but is not necessarily a set of principal axes for that part. We shall assume here that the nature of the elastic motion is such that the mass center of the entire system remains at the origin of $xyz$. In measuring elastic deformations, we consider reference frames $x_iy_iz_i$ fixed relative to the elastic domains $D_i$ $(i = 1, 2, \ldots, n)$, where the direction of these axes is chosen parallel to that of the elastic deformations. The origin of axes $x_iy_iz_i$ is denoted by $O_i$ and in general it need not coincide with $c$.

Next let us denote the radius vector from the mass center $c$ to a point in the domain $D_i$ $(i = 0, 1, \ldots, n)$ by $\rho_i + \xi_i$, where the point coincides with the position of an element of mass $dm_i$ when the body is in undeformed state. The constant-magnitude vector $\rho_i$ denotes the radius vector from $c$ to $O_i$; clearly $\rho_0 = 0$. On the other hand, $\xi_i$ is the radius vector from $O_i$ to the point in question, and its components represent the independent spatial variables associated with a point in the domain $D_i$. Denoting by $i_i$, $j_i$, and $k_i$ the unit vectors along axes $x_i$, $y_i$ and $z_i$, respectively, we can write $\rho_i + \xi_i = (h_x + x_i)i_i + (h_y + y_i)j_i + (h_z + z_i)k_i$ $(i = 0, 1, \ldots, n)$. 5
In describing the elastic deformations, we can use the Lagrangian or the Eulerian approach. According to the Lagrangian approach the independent variables are those of the body in undeformed state, whereas in the Eulerian description of motion the independent variables are those of the body in deformed shape. For infinitesimally small deformations the two points of view coalesce, but for large deformations they do not. When it is necessary to calculate the stresses in a body undergoing large deformations, the Eulerian approach is more convenient. Although we shall be concerned with relatively large deformations, we have no interest in the internal stress distribution, and because of kinematical considerations we shall find it more convenient to use the Lagrangian approach. Hence, denoting by $u_i$ the elastic displacement vector of $dmi$, and recognizing that the vector depends both on spatial position and time, we can write it in the form $u_i = u_i(x_i,y_i,z_i,t)\mathbf{i}_i + v_i(x_i,y_i,z_i,t)\mathbf{j}_i + w_i(x_i,y_i,z_i,t)\mathbf{k}_i$, where $u_i, v_i$ and $w_i$ are displacement components measured along $x_i, y_i$ and $z_i$, respectively. If $R_C$ is the radius vector from the center of force C.F. to the mass center $c$, then the position relative to the inertial space of a mass element $dmi$ in deformed state is given by $R_{di} = R_C + h_i + i_l + u_i$.

It should be noted that, by the definition of the mass center, $\sum_{i=0}^{n} \int_{mi} (h_i + i_l + u_i)dm_i = 0$.

In view of the above discussion, the kinetic energy can be written as

$$ T = \frac{1}{2} \int_{mi} \dot{R}_{di} \cdot \dot{R}_{di} dm_i = \frac{1}{2} m \dot{R}_C \cdot \dot{R}_C + \frac{1}{2} \sum_{i=0}^{n} \int_{mi} (\dot{h}_i + \dot{i}_l + \dot{u}_i) \cdot (\dot{h}_i + \dot{i}_l + \dot{u}_i) dm_i $$

(1)

where the first term on the right side of Eq. (1) is recognized as the kinetic energy of translation of the mass center $c$ and the second one as the kinetic
energy due to motion relative to c. Dots denote derivatives with respect to
time. Denoting by \( \omega \) the angular velocity of the set of axes xyz, hence also
of the sets \( x_iy_iz_i \) (\( i = 1,2,\ldots,n \)), and recalling the expression for the
time derivative of a vector expressed in terms of rotating coordinates, we
obtain

\[
\dot{h}_i + \dot{r}_i + \dot{u}_i = \dot{u}_i + \omega \times (h_i + r_i + u_i)
\]

in which \( \dot{u}_i = u_{i1} + v_{i1} + w_{k1} \) is the velocity of \( dm_i \) relative to c due
to elastic effects alone. Introducing Eq. (2) into (1), we arrive at

\[
T = \frac{1}{2} m \dot{R}_c \cdot \dot{R}_c + \frac{1}{2} \omega \cdot J_d \cdot \omega + \omega \cdot \sum_{i=1}^{n} \int_{m_i} (h_i + r_i + u_i) \times \dot{u}_i \, dm_i \\
+ \frac{1}{2} \sum_{i=1}^{n} \int_{m_i} \dot{u}_i \cdot \dot{u}_i \, dm_i
\]

where \( J_d \) is the inertia dyadic of the body in deformed state about axes
xyz.

Equation (3) is most conveniently expressed in matrix form. The
matrix forms of the vectors \( \dot{R}_c \) and \( \omega \) are simply \( \{\dot{R}_c\} \) and \( \{\omega\} \), respectively.
The inertia dyadic \( J_d \) and the term on the right side of Eq. (3) require
further attention. The inertia dyadic can be written as

\[
J_d = \sum_{i=0}^{n} J_d^{(i)}
\]

where \( J_d^{(i)} \) (\( i = 0,1,\ldots,n \)) is the inertia dyadic associated with domain
\( D_i \) when the corresponding mass is in deformed shape. The superscript
\( i \) indicates that the dyadic is expressed in terms of the base \( x_iy_iz_i \).
This would require that we express \( \omega \) in the same base. It is simpler,
however, to express every \( J_d^{(i)} \) in the base \( xyz \) instead. Denoting the
vector \( r_i \) by \( r_i^{(0)} \) and \( r_i^{(i)} \) when expressed in the base \( xyz \) and \( x_iy_iz_i \),
respectively, and by \( \{r_i^{(0)}\} \) and \( \{r_i^{(i)}\} \) the associated column matrices,
the relation between the two can be written as \( \{r_i^{(0)}\} = \{\varepsilon_i\}^T \{r_i^{(1)}\} \),
where \( \{\varepsilon_i\} \) is the matrix of direction cosines between axes \( x_i y_i z_i \) and \( xyz \). In a similar fashion, if we denote by \( Jd_i^{(0)} \) and \( Jd_i^{(1)} \) the inertia dyadics when expressed in the base \( xyz \) and \( x_i y_i z_i \), respectively, and by \( \{\varepsilon_i\}^{(0)} \) and \( \{\varepsilon_i\}^{(1)} \) the associated inertia matrices, then the relation between the two can be shown to have the form \( \{Jd_i^{(0)}\} = \{\varepsilon_i\}^T \{Jd_i^{(1)}\} [\varepsilon_i] \).

With the understanding that the inertia matrices imply the body in deformed shape, we can drop the subscript \( d \). Moreover, we shall drop the superscript \( i \) when it agrees with the subscript. Hence, the inertia matrix for the entire body, expressed in the base \( xyz \), takes the form \( [J^{(0)}] = \sum_{i=0}^{n} \{\varepsilon_i\}^T [J_i] [\varepsilon_i] \). We note that \( \{\varepsilon_0\} = [1] \), where \( [1] \) is the unit matrix. A similar analysis can be performed with regard to the third term on the right side of Eq. (3). It follows that Eq. (3) can be written in the matrix form

\[
\begin{align*}
T &= \frac{1}{2} m \{R_c\}^T \{R_c\} + \frac{1}{2} \sum_{i=0}^{n} \{\omega_i\}^T [\varepsilon_i]^T [J_i] [\varepsilon_i] \{\omega_i\} + \sum_{i=1}^{n} \int_{m_i} h_i^{(0)} \\
&\quad + r_i^{(0)} + u_i^{(0)} [\varepsilon_i]^T \{\dot{u}_i\} \, dm_i + \frac{1}{2} \sum_{i=1}^{n} \int_{m_i} \{u_i\}^T \{\dot{u}_i\} \, dm_i
\end{align*}
\]

where \( [h_i^{(0)} + r_i^{(0)} + u_i^{(0)}] \) is a skew-symmetric matrix whose elements satisfy the relation \( h_{inn}^{(0)} + r_{inn}^{(0)} + u_{inn}^{(0)} = \sum_{\xi=1}^{3} \epsilon_{nml} (h_{i\xi}^{(0)} + r_{i\xi}^{(0)} + u_{i\xi}^{(0)}) \), in which \( \epsilon_{nml} \) is the epsilon symbol (see Ref. 1, p. 109). Clearly, \( \{u_i\} \) represents the matrix notation of \( \dot{u}_i \).

The potential energy results from two sources, namely, gravity and elastic deformations, denoted by \( V_G \) and \( V_E \), respectively, so that

\[ V = V_G + V_E \]. From Ref. 15, we conclude that the gravitational potential energy can be written as
\[ V_E = \frac{K_m}{R_C} - \frac{K}{2R_C^2} \sum_{i=0}^{n} \text{tr} \left( \{ \xi_i \}^T [J_i] \{ \xi_i \} \right) + \frac{3K}{2R_C^2} \sum_{i=0}^{n} \{ \xi_a \}^T [J_i] [J_i] \{ \xi_a \} \] (5)

where \( \text{tr} \) denotes the trace of a matrix, and \( \{ \xi_a \} \) is the column matrix of direction cosines between the direction of the vector \( \mathbf{R}_C \) and axes \( \mathbf{xyz} \).

The elastic potential energy, also known as strain energy, requires special attention, particularly in the case of large deformations. No general expression, such as for \( T \) and \( V_G \), can be written for \( V_E \). This is so because an explicit form requires the knowledge of the type of elastic members involved. For the moment, we shall be content to write

\[ V_E = \sum_{i=1}^{n} V_{Ei} \] (6)

where \( V_{Ei} \) (\( i = 1, 2, \ldots, n \)) is the elastic potential energy associated with the member occupying the domain \( D_i \) when the member is undeformed. We shall return to the elastic potential energy shortly.

At this point it appears desirable to determine the functional dependence of the kinetic and potential energy in order to derive general Lagrange's equations of motion. To this end, we must specify the nature of the elastic members. We shall be concerned with one-dimensional members capable of flexure in two orthogonal directions. Any axial displacements will be assumed to be a result of change of length caused by the transverse displacements and not because of axial flexibility. In essence, the members are cantilevered bars undergoing large transverse displacements (see Fig. 2). Although we shall use nonlinear theory for the elastic motion, this will be because geometric nonlinearities and not as a result...
of nonlinear stress-strain relations. The mass distribution is arbitrary, but some of the members carry tip masses.

Letting the radius vector \( r_{i} \) be aligned with axis \( x_i \) when the bar is undeformed, we conclude from Fig. 2 that

\[
\frac{r_{i}^{(i)}}{r_{i}} = x_{i}^{(i)} = x_{i}^{(i)}, \quad i = 1, 2, \ldots, n
\]

and

\[
u_{i}^{(i)} = u_{i}(x_{i}, t) = v_{i}(x_{i}, t)_{1} + w_{i}(x_{i}, t)_{2} \quad i = 1, 2, \ldots, n
\]

In view of this, the elements of the inertia matrix for the rigid member can be written as

\[
J_{011} = A_{0}, \quad J_{022} = B_{0}, \quad J_{033} = C_{0}
\]

\[
J_{012} = J_{021} = J_{013} = J_{031} = J_{023} = J_{032} = 0
\]

where \( A_{0}, B_{0}, C_{0} \) are the principal moments of inertia of the rigid part, whereas these for member \( i \) are

\[
J_{i11} = \int_{0}^{\ell_{i}} \rho_{i}[(h_{yi} + v_{i})^{2} + (h_{zi} + w_{i})^{2}]dx_{i} + m_{i}[(h_{yi} + v_{i})^{2} + (h_{zi} + w_{i})^{2}]\bigg|_{x_{i}=\ell_{i}}
\]

\[
J_{i22} = \int_{0}^{\ell_{i}} \rho_{i}[(h_{xi} + x_{i})^{2} + (h_{zi} + w_{i})^{2}]dx_{i} + m_{i}[(h_{xi} + x_{i})^{2} + (h_{zi} + w_{i})^{2}]\bigg|_{x_{i}=\ell_{i}}
\]

\[
J_{i33} = \int_{0}^{\ell_{i}} \rho_{i}[(h_{yi} + v_{i})^{2} + (h_{zi} + w_{i})^{2}]dx_{i} + m_{i}[(h_{yi} + v_{i})^{2} + (h_{zi} + w_{i})^{2}]\bigg|_{x_{i}=\ell_{i}}
\]

\[
J_{i12} = J_{i21} = -\int_{0}^{\ell_{i}} \rho_{i}(h_{xi} + x_{i})(h_{yi} + v_{i})dx_{i} - m_{i}(h_{xi} + x_{i})(h_{yi} + v_{i})\bigg|_{x_{i}=\ell_{i}}
\]

\[
J_{i13} = J_{i31} = -\int_{0}^{\ell_{i}} \rho_{i}(h_{xi} + x_{i})(h_{zi} + w_{i})dx_{i} - m_{i}(h_{xi} + x_{i})(h_{zi} + w_{i})\bigg|_{x_{i}=\ell_{i}}
\]
Note that \( \rho_i \) and \( m_i \) are mass densities and tip masses, respectively, and \( h_{x_i}, h_{y_i}, h_{z_i} \) denote the coordinates of the points of attachment of the booms measured from the mass center along axes \( x_iy_iZ_i \) (\( i = 1,2,\ldots,n \)). We shall not specify the mass densities at this point.

The desired equilibrium configuration is that of gravity-gradient stabilization. That implies that the mass center \( c \) moves in a circular orbit with the constant angular velocity \( \omega \) (see Fig. 3), and the set of axes \( xyz \) coincides with a set of orbital axes \( abc \), where \( a \) coincides with the direction of the radius vector \( R_c \), \( b \) is tangent to the orbit and in the direction of the motion, and \( c \) is normal to the motion. Note that the orbital axes \( abc \) rotate relative to an inertial space with angular velocity \( \omega \) about axis \( c \). The orientation of axes \( xyz \) with respect to \( abc \) is given by three angles \( \theta_j \) and \( \{ \omega \} \) depends on these angles and angular velocities \( \dot{\theta}_j \) (\( j = 1,2,3 \)). Because the first term in the kinetic energy, Eq. (4), is constant for a circular orbit, it will be ignored in future discussions. Moreover, the last term depends on the elastic velocities, so that the functional dependence of \( T \) is

\[
T = T(\theta_j, \dot{\theta}_j, v_i, \dot{v}_i, w_i, \dot{w}_i), \quad j = 1,2,3; \quad i = 1,2,\ldots,n
\]

The gravitational potential energy \( V_G \) contains the matrix \( \{ \xi_a \} \), which is defined as the matrix of direction cosines between \( R_c \) and \( xyz \). Since
xyz can be obtained from abc by means of the rotations $\theta_j$ ($j = 1, 2, 3$), it follows that

$$V_G = V_G (\theta_j, v_i, w_i), \ j = 1, 2, 3; \ i = 1, 2, \ldots, n$$ (12)

It remains to establish the functional dependence of $V_E$. This requires some elaboration, particularly because of the geometric nonlinearities involved. First we wish to distinguish between the potential energy $V_{EA}$ due to axial motion, and the potential energy $V_{EB}$ due to flexure.

Next let us consider Fig. 4 and denote by $s_i$ the distance to any element of mass $dm_i$ when measured along the deflected bar and by $x_i$ when measured along the original direction of the undeflected bar. We shall assume that the bar is inextensional, so that these two distances remain the same, $s_i = x_i$. An element of length along the deflected bar can be obtained from

$$\left( ds_i \right)^2 = \left( dx_i + du_i \right)^2 + (dv_i)^2 + (dw_i)^2$$ (13)

Assuming that $du_i$ is one order of magnitude smaller than $dv_i$ and $dw_i$, recalling that $ds_i = dx_i$, and rearranging Eq. (13), we arrive at

$$du_i = -\frac{1}{2} \left[ \left( \frac{dv_i}{dx_i} \right)^2 + \left( \frac{dw_i}{dx_i} \right)^2 \right] dx_i, \ i = 1, 2, \ldots, n$$ (14)

so that the axial displacement resulting from the transverse displacements is negative. Because for inextensional motion the axial force $P_{xi}$ does not depend on the axial displacement, and, moreover, because a tensile force opposes the motion, we have

$$V_{EA} = -\frac{n}{\Sigma} \int_{\xi} p_{xi} du_i = \frac{1}{2} \frac{n}{\Sigma} \int_{\xi} p_{xi} \left[ \left( \frac{\partial v_i}{\partial x_i} \right)^2 + \left( \frac{\partial w_i}{\partial x_i} \right)^2 \right] dx_i$$ (15)
where total derivatives have been replaced by partial derivatives in recognition of the fact that the displacements are functions not only of spatial position but also of time.

The flexure potential energy is due to the bending displacements \( v_i \) and \( w_i \). We shall denote the bending moment associated with the displacement \( v_i \) by \( M_{z_i} \) and the change in slope corresponding to an element of length in deformed state by \( d\phi_{z_i} \) because they both take place about the \( z_i \)-axis. Accordingly, the analogous quantities associated with flexure about \( y_i \) are denoted by \( M_{y_i} \) and \( d\phi_{y_i} \), respectively. It follows that the flexure potential energy can be written as

\[
V_{EB} = \frac{1}{2} \sum_{i=1}^{n} \int_{z_i} \left( M_{y_i} d\phi_{y_i} + M_{z_i} d\phi_{z_i} \right)
\]  

But the bending moments \( M_{y_i} \) and \( M_{z_i} \) can be written in terms of the associated flexural stiffness and radii of curvature, as follows

\[
M_{y_i} = \frac{E I_{y_i}}{R_{y_i}}, \quad M_{z_i} = \frac{E I_{z_i}}{R_{z_i}}
\]  

where, from Fig. 5, the radii of curvature have the form

\[
R_{y_i} = \frac{d s_{y_i}}{d \phi_{y_i}}, \quad R_{z_i} = \frac{d s_{z_i}}{d \phi_{z_i}}
\]

in which

\[
ds_{z_i} = \left[ 1 + \left( \frac{d v_i}{d x_i} \right)^2 \right]^{1/2} d x_i, \quad ds_{y_i} = \left[ 1 + \left( \frac{d w_i}{d x_i} \right)^2 \right]^{1/2} d x_i
\]

Moreover

\[
\frac{d v_i}{d x_i} = \tan \phi_{z_i}, \quad \frac{d w_i}{d x_i} = \tan \phi_{y_i}
\]
From Eqs. (20), it follows that

\[
d\phi_i = d \left( \tan^{-1} \left( \frac{d\psi_i}{dx_i} \right) \right) = \frac{d \left( \frac{d\psi_i}{dx_i} \right)}{1 + \left( \frac{d\psi_i}{dx_i} \right)^2} = \frac{\frac{d^2\psi_i}{dx_i^2}}{1 + \left( \frac{d\psi_i}{dx_i} \right)^2} \quad \text{dx}_i
\]

Finally, introducing Eqs. (17) through (21) into Eq. (16), we obtain

\[
V_{EB} = \frac{1}{2} \sum_{i=1}^{n} \int_{x_i}^{x_{i+1}} (M_{zi} d\phi_i + M_{yi} d\phi_y) = \frac{1}{2} \sum_{i=1}^{n} \int_{x_i}^{x_{i+1}} \left[ EI_{zi} \frac{(d\phi_i)^2}{ds_{zi}} \right. \\
\left. + EI_{yi} \frac{(d\phi_y)^2}{ds_{yi}} \right] = \frac{1}{2} \sum_{i=1}^{n} \int_{x_i}^{x_{i+1}} \left[ EI_{zi} \frac{(\frac{d^2\psi_i}{dx_i^2})^2}{1 + \left( \frac{d\psi_i}{dx_i} \right)^2} \right. \\
\left. + EI_{yi} \frac{(\frac{d^2\psi_y}{dx_i^2})^2}{1 + \left( \frac{d\psi_y}{dx_i} \right)^2} \right] \text{dx}_i
\]

Recalling that the flexural displacements depend also on time, and writing binomial expansions for the denominators in Eq. (22), we arrive at

\[
V_{EB} = \frac{1}{2} \sum_{i=1}^{n} \int_{x_i}^{x_{i+1}} \left[ EI_{zi} \left( \frac{\partial^2\psi_i}{\partial x_i^2} \right)^2 \left[ 1 - \frac{5}{2} \left( \frac{\partial\psi_i}{\partial x_i} \right)^2 \right] + EI_{yi} \left( \frac{\partial^2\psi_y}{\partial x_i^2} \right)^2 \left[ \frac{1}{2} \left( \frac{\partial\psi_y}{\partial x_i} \right)^2 \right] \\
- \frac{5}{2} \left( \frac{\partial\psi_y}{\partial x_i} \right)^2 \right] \text{dx}_i
\]
where the terms involving $\partial v_i / \partial x_i$ and $\partial w_i / \partial x_i$ are recognized as the corrections due to the geometric nonlinear effect.

In view of the above, the potential energy has the general functional form

$$V_E = V_{EA} + V_{EB}$$

$$= V_E (v_i', v_i'', w_i', w_i''), \quad i = 1, 2, \ldots, n$$

(24)

where primes indicate differentiations with respect to $x_i$.

From Fig. 4, we conclude that we must still account for the distributed forces $p_{yi}$ and $p_{zi}$. Regarding these forces as nonconservative, and assuming that they do not depend on the elastic deformations, we can account for their effect in the form of the nonconservative work

$$W_{nc} = \int_0^{t_2} \left( p_{yi} v_i + p_{zi} w_i \right) dx_i$$

(25)

so that the total work can be written as

$$W = W_c + W_{nc} = -V + W_{nc}$$

(26)

where the conservative work has been recognized as being equal to the negative of the potential energy.

The system differential equations of motion, and the appropriate boundary conditions, can be obtained from the extended Hamilton's principle (see Ref. 1, Sec. 2.7)

$$\int_{t_1}^{t_2} (\delta T + \delta W) dt = 0$$

(27)
where all the virtual displacements must be set equal to zero at $t = t_1, t_2$. Introducing the Lagrangian $L = T - V$, Eq. (27) can be written as

$$
\int_{t_1}^{t_2} (\delta L + \delta W_n) \, dt = 0 \tag{28}
$$

where the Lagrangian has the functional form

$$
L = L(\theta_j, \dot{\theta}_j, v_i, \dot{v}_i, v_i^{(2)}, w_i, \dot{w}_i, w_i^{(2)}) \; , \; j = 1,2,3 \; ; \; i = 1,2,\ldots,n
$$

(29)

It will prove convenient to separate the Lagrangian into that associated with the rigid domain $D_0$ and those associated with the elastic domains $D_i$. Hence, let the Lagrangian have the general functional form (see Ref. 19)

$$
L(t) = L_0(t) + \sum_{i=1}^{n} \left( \int_0^{\xi_i} \hat{L}_i(x_i,t) \, dx_i + L_i(\varepsilon_i,t) \right) \tag{30}
$$

where

$$
L_0(t) = L_0[\theta_j(t), \dot{\theta}_j(t)] , \; \; j = 1,2,3 \tag{31}
$$

$$
\hat{L}_i(x_i,t) = \hat{L}_i[\theta_j(t), \dot{\theta}_j(t), v_i(x_i,t), \dot{v}_i(x_i,t), v_i^{(2)}(x_i,t), w_i(x_i,t), \ldots, w_i^{(2)}(x_i,t)] \; , \; i = 1,2,\ldots,n \tag{32}
$$

$$
L_i(\varepsilon_i,t) = L_i[\theta_j(t), \dot{\theta}_j(t), v_i(\varepsilon_i,t), \dot{v}_i(\varepsilon_i,t), w_i(\varepsilon_i,t), \ldots, w_i^{(2)}(\varepsilon_i,t)] \tag{33}
$$

in which $L_0$ is the Lagrangian corresponding to the system in undeformed state, $\hat{L}_i$ the Lagrangian density associated with any point of the elastic member $i$, and $L_i$ the Lagrangian corresponding to the tip mass. Moreover,
\( \ell_i \) represents the length of member \( i \). From Eqs. (30), (31), and (32), we conclude that

\[
\delta L = \sum_{j=1}^{3} \left( \frac{\partial L}{\partial \dot{\theta}_j} \delta \dot{\theta}_j + \frac{\partial L}{\partial \dot{\theta}_j} \delta \dot{\theta}_j \right) + \sum_{i=1}^{n} \left( \int_{0}^{\ell_i} \left( \frac{\partial \hat{L}_i}{\partial \dot{v}_i} \delta v_i + \frac{\partial \hat{L}_i}{\partial \dot{v}_i} \delta \dot{v}_i \right) + \frac{\partial \hat{L}_i}{\partial \dot{w}_i} \delta w_i \right) dx_i \\
+ \frac{\partial \hat{L}_i}{\partial v_i(\ell_i, t)} \delta v_i(\ell_i, t) + \frac{\partial \hat{L}_i}{\partial v_i(\ell_i, t)} \delta \dot{v}_i(\ell_i, t) + \frac{\partial \hat{L}_i}{\partial w_i(\ell_i, t)} \delta w_i(\ell_i, t) \\
+ \frac{\partial \hat{L}_i}{\partial w_i(\ell_i, t)} \delta \dot{w}_i(\ell_i, t) \right) (33)
\]

In addition,

\[
\delta W_{nc} = \sum_{i=1}^{n} \left( \int_{0}^{\ell_i} \left( p_{yi} \delta v_i + p_{zi} \delta \dot{w}_i \right) dx_i \right) (34)
\]

Inserting Eqs. (33) and (34) into (32), and integrating by parts with respect to \( t \), we arrive at Lagrange's equations for the rotational motion

\[
\frac{\partial L}{\partial \dot{\theta}_j} - \frac{d}{dt} \left( \frac{\partial L}{\partial \dot{\theta}_j} \right) = 0, \quad j = 1, 2, 3 (35)
\]

Moreover, integrating by parts with respect to \( t \) and \( x_i \), we obtain

Lagrange's equations for the transverse displacements, and the associated boundary conditions, in the form

\[
\frac{\partial \hat{L}_i}{\partial v_i} - \frac{\partial \hat{L}_i}{\partial \dot{v}_i} \left( \frac{\partial \hat{L}_i}{\partial \dot{v}_i} \right) - \frac{\partial \hat{L}_i}{\partial x_i} \left( \frac{\partial \hat{L}_i}{\partial x_i} \right) + \frac{\partial \hat{L}_i}{\partial x_i} \left( \frac{\partial \hat{L}_i}{\partial x_i} \right) + p_{yi} = 0, \quad 0 < x_i < \ell_i,
\]

\[
i = 1, 2, \ldots, n (36a)
\]

and

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Equations similar in structure to Eqs. (36) can be written for \( w_i \) by simply replacing \( v_i \) by \( w_i \).

3. Nontrivial Equilibrium

Let us consider the case in which \( p_{yi} = p_{zi} = 0 \) and define an equilibrium configuration as a set of dependent variables \( \theta_j, v_i, w_i \) constant in time and satisfying Lagrange's equations. Because these variables do not depend on time, they must satisfy the equations

\[
\frac{\partial L}{\partial \theta_j} = 0, \quad j = 1, 2, 3
\]  

(37)

and

\[
\frac{\partial \hat{L}_i}{\partial v_i} - \frac{\partial}{\partial x_i} \left( \frac{\partial \hat{L}_i}{\partial v_i'} \right) + \frac{\partial^2}{\partial x_i^2} \left( \frac{\partial \hat{L}_i}{\partial v_i''} \right) = 0, \quad 0 < x_i < \kappa_i, \quad i = 1, 2, \ldots, n
\]

(38a)

\[
\left[ \frac{\partial \hat{L}_i}{\partial v_i} - \frac{\partial}{\partial x_i} \left( \frac{\partial \hat{L}_i}{\partial v_i'} \right) + \frac{\partial L_i}{\partial v_i} \right] \delta v_i = 0, \quad \frac{\partial \hat{L}_i}{\partial v_i'} \delta v'_i = 0 \quad \text{at} \quad x_i = \kappa_i
\]

(38b)

\[i = 1, 2, \ldots, n\]

as well as a set of equations similar to (38) for \( w_i \). We shall denote the solutions of Eqs. (37) and (38), together with the set of equations for \( w_i \),
by \( \theta_{j0}, v_{i0}(x_i), w_{i0}(x_i) \), where the first are constant and the latter functions of the spatial variables \( x_i \) alone.

4. Perturbations About Equilibrium. The Variational Equations of Motion

The interest lies in the stability of the system in the neighborhood of the nontrivial solutions \( \theta_{j0}, v_{i0}(x_i), w_{i0}(x_i) \). We shall seek stability criteria by means of Liapunov's direct method, and, to this end, we let the solutions of Eqs. (35) and (36) and the companion equations to (36) have the form

\[
\theta_j(t) = \theta_{j0} + \theta_{j1}(t), \quad j = 1,2,3
\]

\[
v_i(x_i,t) = v_{i0}(x_i) + v_{i1}(x_i,t), \quad w_i(x_i,t) = w_{i0}(x_i) + w_{i1}(x_i,t),
\]

where \( \theta_{j1}(t), v_{i1}(x_i,t), w_{i1}(x_i,t) \) are small perturbations. Inserting Eqs. (39) into Eq. (30), and expanding a Taylor's series about the nontrivial equilibrium, we obtain

\[
L = L(\theta_{j0}, v_{i0}, v_{i0}'', w_{i0}, w_{i0}'', w_{i0}'') + \sum_{j=1}^{3} \left( \frac{\partial L}{\partial \theta_{j0}} \theta_{j0} + \frac{\partial L}{\partial \theta_{j0}} \theta_{j1} \right)
\]

\[
+ \sum_{i=1}^{n} \left[ \int_0^t \left( \frac{\partial L}{\partial v_{i0}'} v_{i1} + \frac{\partial L}{\partial v_{i0}''} v_{i1}' + \frac{\partial L}{\partial w_{i0}'} w_{i1}' + \frac{\partial L}{\partial w_{i0}''} w_{i1}'' \right) dx_i + \left( \frac{\partial L}{\partial v_{i0}'} v_{i1} + \ldots + \frac{\partial L}{\partial w_{i0}'} w_{i1} \right) \right]_{x_i = \xi_i}
\]

\[
+ \frac{1}{2} \sum_{j=1}^{3} \sum_{k=1}^{3} \left( \frac{\partial^2 L}{\partial \theta_{j0} \partial \theta_{k0}} \theta_{j1} \theta_{k1} + 2 \frac{\partial^2 L}{\partial \theta_{j0} \partial \theta_{k0}} \theta_{j1} \theta_{k1} + \frac{\partial^2 L}{\partial \theta_{j0} \partial \theta_{k0}} \theta_{j1} \theta_{k1} \right)
\]
\[ + \frac{1}{2} \sum_{i=1}^{n} \left[ \int_{0}^{l} \left( \frac{\partial L_i}{\partial v_i} \dot{v}_{i1}^2 + \frac{\partial^2 L_i}{\partial v^2_i} \ddot{v}_{i1}^2 + \frac{\partial^2 L_i}{\partial v_{12}^2} \dddot{v}_{i1}^2 + \frac{\partial^2 L_i}{\partial w_{12}^2} \dddot{w}_{i1}^2 \right) dx_1 \right] \]

\[ + \frac{3}{2} \sum_{j=1}^{n} \left[ \frac{\partial L_i}{\partial \theta_j} \dot{\theta}_{j1} \right] \]

where \( \frac{\partial L_i}{\partial \theta_j} \) = \( \frac{\partial L_i}{\partial \theta_j} \) \( \theta_j = \theta_j, \ldots \)

etc. But the term \( L(\theta_j, v_{i1}, \ldots, w_{i1}^n) \) is constant. Moreover, by virtue of Eqs. (37) and (38) and the companion equations for \( w_i \), all the linear terms in the perturbed variables in expansion (40) reduce to

\[ \frac{3}{2} \sum_{j=1}^{n} \left[ \frac{\partial L_i}{\partial \theta_j} \dot{\theta}_{j1} + \sum_{i=1}^{n} \left[ \int_{0}^{l} \left( \frac{\partial L_i}{\partial v_i} \dot{v}_{i1} + \frac{\partial L_i}{\partial w_{i1}} \dot{w}_{i1} \right) dx_1 \right] + \left( \frac{\partial L_i}{\partial v_{i1}} \right) \]
which are all linear in the generalized velocities $\hat{\phi}_{j1}, \dot{v}_{i1}, \ddot{w}_{i1}$. In view of this, if we retain terms through second order only, the Lagrangian becomes

$$L = T_{21} + T_{11} + T_{01} - V_1$$

where

$$T_{21} = \frac{1}{2} \sum_{j=1}^{3} \sum_{k=1}^{3} \left( \frac{\partial^2 L}{\partial \dot{\phi}_{j0} \partial \dot{\phi}_{k0}} \cdot \dot{\phi}_{j1} \dot{\phi}_{k1} \right) + \frac{1}{2} \sum_{i=1}^{n} \sum_{j=1}^{3} \left( \frac{\partial^2 L}{\partial \dot{\phi}_{j0} \partial \dot{\phi}_{i0}} \cdot \dot{\phi}_{j1} \dot{\phi}_{i1} \right) \right) \frac{dx_i}{dx_i}$$

$$+ 2 \left[ \frac{\partial^2 L}{\partial \dot{v}_{i1} \partial \dot{w}_{i1}} \cdot \dot{v}_{i1} \dot{w}_{i1} + \sum_{j=1}^{3} \left( \frac{\partial^2 L}{\partial \dot{v}_{i1} \partial \dot{v}_{j1}} \cdot \dot{v}_{i1} \dot{v}_{j1} \right) \right]$$

$$+ \left[ \frac{\partial^2 L}{\partial \ddot{v}_{i1}} \cdot \ddot{v}_{i1} + \sum_{j=1}^{3} \left( \frac{\partial^2 L}{\partial \ddot{v}_{i1} \partial \dot{v}_{j1}} \cdot \ddot{v}_{i1} \dot{v}_{j1} \right) \right]$$

$$+ \sum_{j=1}^{3} \left( \frac{\partial^2 L}{\partial \dot{w}_{i1} \partial \dot{v}_{j1}} \cdot \dot{w}_{i1} \dot{v}_{j1} \right) \right) \right) \frac{dx_i}{dx_i}$$

is quadratic in the generalized velocities,

$$T_{11} = \sum_{j=1}^{3} \sum_{k=1}^{3} \left( \frac{\partial^2 L}{\partial \dot{\phi}_{j0} \partial \dot{\phi}_{k0}} \cdot \dot{\phi}_{j1} \dot{\phi}_{k1} \right) + \sum_{i=1}^{n} \sum_{j=1}^{3} \left( \frac{\partial^2 L}{\partial \dot{\phi}_{j0} \partial \dot{\phi}_{i0}} \cdot \dot{\phi}_{j1} \dot{\phi}_{i1} \right)$$

$$+ \sum_{j=1}^{3} \left( \frac{\partial^2 L}{\partial \dot{v}_{i1} \partial \dot{v}_{j1}} \cdot \dot{v}_{i1} \dot{v}_{j1} \right) + \sum_{j=1}^{3} \left( \frac{\partial^2 L}{\partial \dot{w}_{i1} \partial \dot{v}_{j1}} \cdot \dot{w}_{i1} \dot{v}_{j1} \right)$$

$$+ \sum_{j=1}^{3} \left( \frac{\partial^2 L}{\partial \dot{w}_{i1} \partial \dot{w}_{j1}} \cdot \dot{w}_{i1} \dot{w}_{j1} \right) \right)$$

$$\left. \frac{dx_i}{dx_i} \right|_{x_i = \ell_i}$$

$$(41)$$

$$(42)$$

$$(43)$$
is linear in the generalized velocities, and

\[
T_{01} - V_1 = \frac{1}{2} \sum_{j=1}^{3} \sum_{k=1}^{3} \frac{\partial^2 L}{\partial q_j \partial q_k} \theta_{j1} \theta_{k1} + \frac{1}{2} \sum_{i=1}^{n} \left\{ \frac{\partial^2 L}{\partial \dot{q}_i^2} \dot{q}_i^2 + \frac{\partial^2 L}{\partial \dot{q}_i \partial \dot{q}_i} \dot{q}_i \right\} + \sum_{j=1}^{3} \left( \frac{\partial^2 L}{\partial \theta_{j1} \partial \theta_{j1}} \theta_{j1} \dot{q}_i^2 \right) + 2 \left( \frac{\partial^2 L}{\partial \theta_{j1} \partial \dot{q}_i} \theta_{j1} \dot{q}_i \dot{q}_i + \frac{\partial^2 L}{\partial \theta_{j1} \partial \ddot{q}_i} \theta_{j1} \ddot{q}_i \right) + 2 \left( \frac{\partial^2 L}{\partial \dot{q}_i \partial \ddot{q}_i} \dot{q}_i \ddot{q}_i + \frac{\partial^2 L}{\partial \dot{q}_i \partial \dot{q}_i} \dot{q}_i \dot{q}_i \right) + 2 \sum_{j=1}^{3} \left( \frac{\partial^2 L}{\partial \theta_{j1} \partial \dot{q}_i} \theta_{j1} \dot{q}_i \right) \right\}
\]

is free of generalized velocities.

In view of the above, the perturbed Lagrangian can be written in the general functional term

\[
L = L(\theta_{j1}, \dot{\theta}_{j1}, \dot{v}_{i1}, \ddot{v}_{i1}, v_{i1}, ..., \dddot{w}_{i1}, \dddot{w}_{i1}), \; j = 1, 2, 3; \; i = 1, 2, ..., n
\]
Consequently, the variational equations can be written in the form of the Lagrange equations, Eqs. (35) and (36), but with the subscripts $j$ and $i$ replaced by $jl$ and $il$, respectively. Unlike Eqs. (35) and (36), the variational equations possess trivial equilibrium.

5. Discretization by a Rayleigh-Ritz Approach

The variational equations discussed in the preceding section constitute a set of hybrid differential equations, in the sense that the equations for the rotational motion are ordinary differential equations and those for the elastic displacements are partial differential equations, where the latter are subject to given boundary conditions. It will prove convenient to transform the system into one consisting of ordinary differential equations alone. This can be done by using a discretization procedure based on the Rayleigh-Ritz approach. Indeed, let us introduce the notation

\[ \theta_{jl}(t) = q_{j}(t), \quad j = 1,2,3 \]

\[ v_{11}(x_1,t) = \sum_{j=4}^{p+3} \phi_{j}(x_1)q_{j}(t), \quad w_{11}(x_1,t) = \sum_{j=p+4}^{2p+3} \psi_{j}(x_1)q_{j}(t) \]

\[ v_{21}(x_2,t) = \sum_{j=2p+4}^{3p+3} \phi_{j}(x_2)q_{j}(t), \quad w_{21}(x_2,t) = \sum_{j=3p+4}^{4p+3} \psi_{j}(x_2)q_{j}(t) \]

\[ v_{n1}(x_n,t) = \sum_{j=(2n-1)p+4}^{(2n-1)p+3} \phi_{j}(x_n)q_{j}(t), \quad w_{n1}(x_n,t) = \sum_{j=(2n-1)p+4}^{2np+3} \psi_{j}(x_n)q_{j}(t) \]

where $\phi_{j}(x_1)$ and $\psi_{j}(x_1)$ are admissible functions, taken as the eigenfunctions of the linearized system. With this notation, Eq. (43) can
be written in the matrix form

\[ T_{21} = \frac{1}{2} \{ \dot{q}(t) \}^T [m] \{ \dot{q}(t) \} \]  

(48)

where \([m]\) is a constant symmetric matrix having the elements

\[ m_{jk} = \frac{\partial^2 L}{\partial \dot{q}_i \partial \dot{q}_k}, \quad j, k = 1, 2, 3 \]  

(49a)

\[ m_{jk} = \int_0^{2\pi} \frac{\partial^2 \hat{L}_i}{\partial \dot{q}_i \partial \dot{q}_k} \phi_k(x_i) dx_i + \left[ \frac{\partial^2 L_i}{\partial \dot{q}_i \partial \dot{q}_k} \phi_k(x_i) \right] \quad x_i = \hat{x}_i \]  

\[ j = 1, 2, 3; \quad k = 2(i-1)p+4, 2(i-1)p+5, \ldots, (2i-1)p+3, \quad i = 1, 2, \ldots, n \]  

(49b)

\[ m_{jk} = \int_0^{2\pi} \frac{\partial^2 \hat{L}_i}{\partial \dot{q}_i \partial \dot{q}_k} \psi_k(x_i) dx_i + \left[ \frac{\partial^2 L_i}{\partial \dot{q}_i \partial \dot{q}_k} \psi_k(x_i) \right] \quad x_i = \hat{x}_i \]  

\[ j = 1, 2, 3; \quad k = (2i-1)p+4, (2i-1)p+5, \ldots, 2ip+3, \quad i = 1, 2, \ldots, n \]  

(49c)

\[ m_{jk} = \int_0^{2\pi} \frac{\partial^2 \hat{L}_i}{\partial \dot{q}_i \partial \dot{q}_k} \phi_j(x_i) \phi_k(x_i) dx_i + \left[ \frac{\partial^2 L_i}{\partial \dot{q}_i \partial \dot{q}_k} \phi_j(x_i) \phi_k(x_i) \right] \quad x_i = \hat{x}_i \]  

\[ j, k = 2(i-1)p+4, 2(i-1)p+5, \ldots, (2i-1)p+3, \quad i = 1, 2, \ldots, n \]  

(49d)

\[ m_{jk} = \int_0^{2\pi} \frac{\partial^2 \hat{L}_i}{\partial \dot{q}_i \partial \dot{q}_k} \psi_j(x_i) \psi_k(x_i) dx_i + \left[ \frac{\partial^2 L_i}{\partial \dot{q}_i \partial \dot{q}_k} \psi_j(x_i) \psi_k(x_i) \right] \quad x_i = \hat{x}_i \]  

\[ j = 2(i-1)p+4, 2(i-1)p+5, \ldots, (2i-1)p+3, \quad i = 1, 2, \ldots, n \]  

(49e)

\[ k = (2i-1)p+4, (2i-1)p+5, \ldots, 21p+3, \quad i = 1, 2, \ldots, n \]
\[ m_{jk} = \int_0^{\xi_i} \frac{a^2 L_i}{\partial \dot{w}_{i0} \partial^2 \psi_j(x_i)dx_i + \frac{\partial L_i}{\partial \dot{w}_{i0}} \psi_j(x_i) dx} \left[ \frac{\partial^2 L_i}{\partial \dot{w}_{i0}^2} \psi_k(x_i) \right] x_i = \varepsilon_i \]

\[ j, k = (2i-1)p+4, (2i-1)p+5, \ldots, 2ip+3, i = 1, 2, \ldots, n \]  

(49f)

On the other hand, Eq. (44) leads to the matrix form

\[ T_{11} = \{q(t)\}^T [f] \{\dot{q}(t)\} \]  

(50)

where \([f]\) is a constant square matrix with the elements

\[ f_{jk} = \frac{a^2 L_i}{\partial \dot{w}_{i0} \partial^2 \phi_k} \]  

\[ j, k = 1, 2, 3 \]  

(51a)

\[ f_{jk} = \int_0^{\xi_i} \frac{a^2 L_i}{\partial \dot{w}_{i0} \partial^2 \phi_k(x_i) dx_i + \frac{\partial L_i}{\partial \dot{w}_{i0}} \phi_k(x_i) dx} \left[ \frac{\partial^2 L_i}{\partial \dot{w}_{i0}^2} \phi_k(x_i) \right] x_i = \varepsilon_i \]

\[ j = 1, 2, 3; k = 2(i-1)p+4, 2(i-1)p+5, \ldots, (2i-1)p+3, i = 1, 2, \ldots, n \]  

(51b)

\[ f_{jk} = \int_0^{\xi_i} \frac{a^2 L_i}{\partial \dot{k}_0 \partial^2 \psi_j(x_i) dx_i + \frac{\partial L_i}{\partial \dot{k}_0} \psi_j(x_i) dx} \left[ \frac{\partial^2 L_i}{\partial \dot{k}_0^2} \psi_k(x_i) \right] x_i = \varepsilon_i \]

\[ j = 1, 2, 3; k = (2i-1)p+4, (2i-1)p+5, \ldots, 2ip+3, i = 1, 2, \ldots, n \]  

(51c)

\[ f_{jk} = \int_0^{\xi_i} \frac{a^2 L_i}{\partial \dot{v}_{i0} \partial^2 \phi_j(x_i) dx_i + \frac{\partial L_i}{\partial \dot{v}_{i0}} \phi_j(x_i) dx} \left[ \frac{\partial^2 L_i}{\partial \dot{v}_{i0}^2} \phi_j(x_i) \right] x_i = \varepsilon_i \]

\[ j = 2(i-1)p+4, 2(i-1)p+5, \ldots, (2i-1)p+3, i = 1, 2, \ldots, n; \]

\[ k = 1, 2, 3 \]  

(51d)
\[ f_{jk} = \int_0^{\ell_i} \frac{2L_i}{\partial k_0 \partial w_{i0}} \psi_j(x_i) dx_i + \left[ \frac{2L_i}{\partial k_0 \partial w_{i0}} \phi_j(x_i) \right] x_i = \ell_i \]

\[ j = (2i-1)p+4, (2i-1)p+5, \ldots, 2ip+3, i = 1, 2, \ldots, n; k = 1, 2, 3 \quad (51e) \]

\[ f_{jk} = \int_0^{\ell_i} \frac{2L_i}{\partial v_{i0} \partial w_{i0}} \phi_j(x_i) \psi_k(x_i) dx_i + \left[ \frac{2L_i}{\partial v_{i0} \partial w_{i0}} \phi_j(x_i) \psi_k(x_i) \right] x_i = \ell_i \]

\[ j = 2(i-1)p+4, 2(i-1)p+5, \ldots, (2i-1)p+3, i = 1, 2, \ldots, n \quad (51f) \]

\[ f_{jk} = \int_0^{\ell_i} \frac{2L_i}{\partial v_{i0} \partial w_{i0}} \phi_k(x_i) \psi_j(x_i) dx_i + \left[ \frac{2L_i}{\partial v_{i0} \partial w_{i0}} \phi_k(x_i) \psi_j(x_i) \right] x_i = \ell_i \]

\[ j = (2i-1)p+4, (2i-1)p+5, \ldots, 2ip+3, i = 1, 2, \ldots, n \quad (51g) \]

Finally, from Eq. (45), we can write

\[ T_{01} - V_1 = -\frac{1}{2} [q(t)]^T [k] [q(t)] \quad (52) \]

where \([k]\) is a constant symmetric matrix with the elements

\[ k_{jk} = -\frac{a_{L}}{a_{\theta_{j0}} a_{\theta_{k0}}}, j, k = 1, 2, 3 \quad (53a) \]

\[ k_{jk} = -\int_0^{\ell_i} \frac{2L_i}{\partial \theta_{j0} \partial \theta_{k0}} \phi_k(x_i) dx_i + \left[ \frac{2L_i}{\partial \theta_{j0} \partial \theta_{k0}} \phi_k(x_i) \right] x_i = \ell_i \]

\[ j = 1, 2, 3; k = 2(i-1)p+4, 2(i-1)p+5, \ldots, (2i-1)p+3, i = 1, 2, \ldots, n \quad (53b) \]
\[ k_{jk} = -\int_{0}^{\xi_i} \left[ \frac{a^2 \hat{L}_i}{\partial \psi_k} \psi_k(x_i) \frac{\partial}{\partial x_i} - \left[ \frac{a^2 \hat{L}_i}{\partial \psi_k} \psi_k(x_i) \right] \right] x_i = \varepsilon_i \]

\[ j = 1, 2, 3; k = (2i-1)p+4, (2i-1)p+5, \ldots, 2ip+3, i = 1, 2, \ldots, n \] (53c)

\[ k_{jk} = -\int_{0}^{\xi_i} \left\{ \frac{a^2 \hat{L}_i}{\partial x^2} \frac{\partial}{\partial x^2} + \frac{a^2 \hat{L}_i}{\partial x^1} \frac{\partial}{\partial x^1} \right\} \left[ \phi_j(x_i) \frac{\partial^2}{\partial x^2} \phi_k(x_i) + \frac{a^2 \hat{L}_i}{\partial x^1} \phi_j(x_i) \frac{\partial}{\partial x^1} \phi_k(x_i) \right] \]

\[ + \frac{a^2 \hat{L}_i}{\partial x^1 \partial x^2} \left[ \phi_j(x_i) \frac{\partial^2}{\partial x^2} \phi_k(x_i) + \phi_j(x_i) \frac{\partial}{\partial x^1} \phi_k(x_i) \right] \right\} dx_i \]

\[ - \left[ \frac{a^2 \hat{L}_i}{\partial x^1} \phi_j(x_i) \frac{\partial}{\partial x^1} \phi_k(x_i) \right] x_i = \varepsilon_i \]

\[ j, k = 2(i-1)p+4, 2(i-1)p+5, \ldots, (2i-1)p+3, i = 1, 2, \ldots, n \] (53d)

\[ k_{jk} = -\int_{0}^{\xi_i} \left[ \frac{a^2 \hat{L}_i}{\partial x^1} \phi_j(x_i) \psi_k(x_i) \frac{\partial}{\partial x_i} - \left[ \frac{a^2 \hat{L}_i}{\partial x^1} \phi_j(x_i) \psi_k(x_i) \right] \right] x_i = \varepsilon_i \]

\[ j = 2(i-1)p+4, 2(i-1)p+5, \ldots, (2i-1)p+3, i = 1, 2, \ldots, n \] (53e)

\[ k = (2i-1)p+4, (2i-1)p+5, \ldots, 2ip+3 \]
$$-\left[\frac{3^2L_i}{2}\psi_j(x_i)\psi_k(x_i)\right]_{x_i} = \xi_i$$

$$j,k = (2i-1)p+4, (2i-1)p+5, \ldots, 2ip+3, i = 1,2,\ldots,n \quad (53f)$$

Introducing Eqs. (48), (50), and (52) into Eq. (42), we can write the Lagrangian in the matrix form

$$L = \frac{1}{2}\{\dot{q}\}^T[m]\{\dot{q}\} + \{q\}^T[f]\{\dot{q}\} - \frac{1}{2}\{q\}^T[k]\{q\} \quad (54)$$

Using the approach of Ref. 21 (see Sec. 3-4), we can write Lagrange's equations in the matrix form

$$\frac{d}{dt} \left\{\frac{\partial L}{\partial \dot{q}}\right\} - \left\{\frac{\partial L}{\partial q}\right\} = \{0\} \quad (55)$$

Hence inserting Eq. (54) into (55), we obtain the equations of motion

$$[m]\{\ddot{q}\} + ([f]^T - [f])\{\dot{q}\} + [k]\{q\} = \{0\} \quad (56)$$

so that, introducing the notation

$$[g] = [f]^T - [f] \quad (57)$$

where $[g]$ is a skew-symmetric matrix, $[g]^T = -[g]$, we obtain

$$[m]\{\ddot{q}\} + [g]\{\dot{q}\} + [k]\{q\} = \{0\} \quad (58)$$

where $[m]$ is identified as the inertia matrix, $[g]$ is a "gyroscopic" matrix and $[k]$ is a stiffness matrix which includes terms due to elastic, gravitational, and centrifugal effects.
6. **Liapunov Stability Analysis**

We shall seek criteria for the stability of motion in the neighborhood of the nontrivial equilibrium by means of the Liapunov direct method. This is equivalent to the problem of stability of the perturbed motion about the trivial solution. In terms of the discretized system, the perturbed motion is described by the vector \( \{q(t)\} \), so that the interest lies in a stability analysis about the trivial equilibrium \( \{q\} = \{0\} \).

It was shown in Ref. 15 that the Hamiltonian is a suitable Liapunov function for the type of problem at hand. Assuming that the system possesses a certain amount of internal damping, however small, the equilibrium is asymptotically stable if the Hamiltonian is positive definite. In terms of the perturbed variables, the Hamiltonian has the form

\[
H = T_{21} - T_{01} + V_1 = \frac{1}{2}\{\dot{q}(t)\}^T[m]\{\dot{q}(t)\} + \frac{1}{2}\{q(t)\}^T[k]\{q(t)\}
\]  

(59)

But the function \( T_{21} \) is positive definite in the generalized velocities \( \dot{q}_j(t) \) by definition. Hence, if the function

\[
\kappa = \frac{1}{2}\{q(t)\}^T[k]\{q(t)\}
\]

is positive definite in the generalized coordinates \( q_j(t) \), then the Hamiltonian is a positive function in the generalized coordinates and velocities and the equilibrium is asymptotically stable. The function \( \kappa \) is positive definite if the matrix \([k]\) is positive definite. Whether \([k]\) is positive definite or not can be ascertained by means of Sylvester's criterion (Ref. 1, Sec. 6.7). The matrix \([k]\) will be referred to as a Hessian matrix.
7. Natural Frequencies of the Complete Structure

The Liapunov direct method provides qualitative information concerning the stability or lack of stability of an equilibrium configuration. Similar information can be extracted from the system of equations (58) via the eigenvalues. In addition, the eigenvalue problem yields results of a more quantitative nature in the form of the system natural frequencies and the normal modes for the complete structure, where the latter are defined later. It turns out that Eqs. (58) lead to an eigenvalue problem of a special nature. The nature of the eigenvalue problem can be conveniently discussed by converting the set of equations from second order to first order. Indeed, if the configuration vector \( \{q(t)\} \) is of dimension \( N \), then we can introduce the \( 2N \)-dimensional state vector \( \{x(t)\} \) in the form

\[
\{x(t)\} = \begin{pmatrix} \dot{\{q(t)\}} \\ \{q(t)\} \end{pmatrix}
\]

No confusion should arise from denoting the state vector by \( \{x(t)\} \), because the symbol \( x_i \) used to denote the position of a point in the elastic members represents a spatial coordinate independent of time and not a time-dependent generalized coordinate. Accordingly, if we introduce the \( 2N \times 2N \) matrices

\[
[M] = \begin{bmatrix} [m] & [0] \\ [0] & [k] \end{bmatrix}, \quad [G] = \begin{bmatrix} [g] & [k] \\ -[k] & [0] \end{bmatrix}
\]

then the set of \( N \) equations (58) can be transformed into a set of \( 2N \) first-order equations having the matrix form.
\[ [M] \{ \dot{x}(t) \} + [G] \{ x(t) \} = \{ 0 \} \]  \hspace{1cm} (63)

where \([M]\) is symmetric and \([G]\) is skew-symmetric,

\[ [M] = [M]^T, \quad [G] = -[G]^T \]  \hspace{1cm} (64)

because \([m]\) and \([k]\) are symmetric and \([g]\) is skew-symmetric.

The matrix equation (63) is of the special form treated in Ref. 20, so that the eigenvalue problem can be solved by the method developed there. Hence, letting

\[ \{ x(t) \} = e^{\lambda t} \{ x \} \]  \hspace{1cm} (65)

where \(\lambda\) and \(\{ x \}\) are constant, we obtain the eigenvalue problem

\[ \lambda \{ M \} \{ x \} + \{ G \} \{ x \} = \{ 0 \} \]  \hspace{1cm} (66)

It is shown in Ref. 20 that the solution of the eigenvalue problem (66) consists of \(2N\) eigenvalues \(\lambda_r\) and eigenvectors \(\{ x \}_r (r = 1, 2, ..., 2N)\), where the eigenvalues consist of pairs of pure imaginary complex conjugates, \(\lambda_r = \pm \omega_r i\), and the eigenvectors also consist of pairs of associated complex conjugates \(\{ x \}_r\) and \(\{ x^* \}_r (r = 1, 2, ..., N)\). Moreover, the eigenvectors are orthogonal in a certain sense. Reference 20 provides an algorithm whereby the eigenvalue problem can be solved in terms of real quantities. The method will be used later in this work to solve the eigenvalue problem for a specific spacecraft.

8. **Lagrange's Equations in Explicit Form**

Lagrange's equations, Eqs. (35) and (36), are written in a general form. Before obtaining the nontrivial equilibrium and the
corresponding variational equations, we must express them in a form in which the various coordinates appear explicitly. By virtue of the assumption that the satellite mass center moves in a circular orbit with orbital velocity \( \Omega \), we can replace \( K/R_c^3 \) by \( \Omega^2 \) in Eq. (5). Moreover, the first terms in Eqs. (4) and (5) can be ignored because they are constant. In view of this, if we recall that the Lagrangian can be written as

\[
L = T - V_G - V_{EA} - V_{EB},
\]

then we can substitute Eqs. (4), (5), (15), and (23) into \( L \), and obtain

\[
L(t) = \frac{1}{2} \{\omega\}^T [J(0)] \{\omega\} + \{\omega\}^T \{K\} + T_E + \frac{1}{2} \Omega^2 \text{tr}[J(0)]
\]

\[
- \frac{3}{2} \Omega^2 \{\varepsilon_a\}^T [J(0)] \{\varepsilon_a\} - \frac{1}{2} \sum_{i=1}^{n} \int_0^{\varepsilon_i} [P_{x_i} (v_i^2 + w_i^2) + E I_z i v_i^2 (1 - \frac{5}{2} v_i^2) + E I_y w_i^2 (1 - \frac{5}{2} w_i^2)] dx_i
\]

where \( P_{x_i} \) is the axial force at any point of the slender rod, and

\[
[J(0)] = \sum_{i=0}^{n} [J_i(0)] = \sum_{i=0}^{n} [\varepsilon_i]^T [J_i][\varepsilon_i]
\]

\[
[K] = \sum_{i=1}^{n} \left( \int_0^{\varepsilon_i} \rho_i [h_i(0) + r_i(0) + u_i(0)] [\varepsilon_i]^T [\dot{\varepsilon_i}] dx_i + m_i [h_i(0) + r_i(0) + u_i(0)] [\varepsilon_i]^T [\dot{\varepsilon_i}] \right)
\]

\[
T_E = \frac{1}{2} \sum_{i=1}^{n} \left( \int_0^{\varepsilon_i} \rho_i [\dot{\varepsilon_i}]^T [\ddot{\varepsilon_i}] dx_i + m_i [\dot{\varepsilon_i}]^T [\ddot{\varepsilon_i}] \right)
\]

(68a)
in which \([J(0)]\) is the inertia matrix of the body in deformed state in terms of the reference system \(xyz\), \((K)\) is an angular momentum matrix due to the elastic velocities, and \(T_E\) is the kinetic energy due to the elastic velocities. The elements of \([J_i]\) are given by Eqs. (9) and (10). Introducing the notation

\[
\hat{J}_{i11} = \rho_i [(h_{yi} + v_i)^2 + (h_{zi} + w_i)^2], \quad J_{i11}(\varepsilon_i) = m_i [(h_{yi} + v_i)^2 + (h_{zi} + w_i)^2] \bigg|_{x_i = \varepsilon_i}
\]

\[
\hat{J}_{i22} = \rho_i [(h_{xi} + x_i)^2 + (h_{zi} + w_i)^2], \quad J_{i22}(\varepsilon_i) = m_i [(h_{xi} + x_i)^2 + (h_{zi} + w_i)^2] \bigg|_{x_i = \varepsilon_i}
\]

\[
\hat{J}_{i33} = \rho_i [(h_{xi} + x_i)^2 + (h_{yi} + v_i)^2], \quad J_{i33}(\varepsilon_i) = m_i [(h_{xi} + x_i)^2 + (h_{yi} + v_i)^2] \bigg|_{x_i = \varepsilon_i}
\]

\[
\hat{J}_{i12} = \hat{J}_{i21} = -\rho_i (h_{xi} + x_i)(h_{yi} + v_i), \quad J_{i12}(\varepsilon_i) = J_{i21}(\varepsilon_i) = -m_i (h_{xi} + x_i)(h_{yi} + v_i) \bigg|_{x_i = \varepsilon_i}
\]

\[
\hat{J}_{i13} = \hat{J}_{i31} = -\rho_i (h_{xi} + x_i)(h_{zi} + w_i), \quad J_{i13}(\varepsilon_i) = J_{i31}(\varepsilon_i) = -m_i (h_{xi} + x_i)(h_{zi} + w_i) \bigg|_{x_i = \varepsilon_i}
\]

\[
\hat{J}_{i23} = \hat{J}_{i32} = -\rho_i (h_{yi} + v_i)(h_{zi} + w_i), \quad J_{i23}(\varepsilon_i) = J_{i32}(\varepsilon_i) = -m_i (h_{yi} + v_i)(h_{zi} + w_i) \bigg|_{x_i = \varepsilon_i}
\]

we can write

\[
[J_i] = \int_0^{\varepsilon_i} [\hat{J}_i] dx_i + [J_i(\varepsilon_i)]
\]

(69)

(70)
In a similar way, from the second of Eqs. (68), we have

\[
\{K\} = \sum_{i=1}^{n} \left[ \int_{0}^{\ell_i} \{K_i\} \, dx_i + \{K_i(\ell_i)\} \right]
\]  

(71)

In view of the above, the Lagrangian densities can be written as follows

\[
\hat{L}_i(x_i,t) = \frac{1}{2} \{\omega\}^T [\hat{J}_i^{(0)}] \{\omega\} + \{\omega\}^T \{\hat{K}_i\} + \frac{1}{2} \rho_i \{\hat{u}_i\}^T \{\hat{u}_i\} + \frac{1}{2} \Omega^2 \text{tr}[\hat{J}_i^{(0)}]
\]

\[
- \frac{3}{2} \Omega^2 \{\ell_a\}^T [\hat{J}_i^{(0)}] \{\ell_a\} - \frac{1}{2} P_{x_i} (v_i^2 + w_i^2) - \frac{1}{2} EZ_{z_i} v_i^2 (1 - \frac{5}{2} w_i^2)
\]

\[
- \frac{1}{2} EI_{y_i} w_i^2 (1 - \frac{5}{2} w_i^2), \quad i = 1,2,\ldots,n
\]

(72)

whereas the parts of the Lagrangian associated with the discrete masses are

\[
L_i(\ell_i,t) = \frac{1}{2} \{\omega\}^T [J_i^{(0)}(\ell_i)] \{\omega\} + \{\omega\}^T \{K_i(\ell_i)\} + \frac{1}{2} m_i \{\hat{u}_i\}^T \{\hat{u}_i\} \bigg|_{x_f = \ell_i}
\]

\[
+ \frac{1}{2} \Omega^2 \text{tr}[J_i^{(0)}(\ell_i)] - \frac{3}{2} \Omega^2 \{\ell_a\}^T [J_i^{(0)}(\ell_i)] \{\ell_a\}
\]

(73)

From the context it should be obvious when brackets and braces denote matrices in Eqs. (67) through (73) and when they do not.

Substituting Eq. (67) into Lagrange's equations for the rotational motion, Eqs. (55), we obtain

\[
\left[ \frac{\partial}{\partial t} \{\omega\}^T \right] [J^{(0)}] \{\omega\} + \left[ \frac{\partial}{\partial \ell_j} \{\omega\}^T \right] \{\ell_j\} - 3 \Omega^2 \left( \frac{\partial}{\partial \ell_j} \{\ell_a\}^T \right) [J^{(0)}] \{\ell_a\}
\]

\[
- \frac{\partial}{\partial t} \left( \frac{\partial}{\partial \ell_i} \{\omega\}^T \right) ([J^{(0)}] \{\omega\} + \{K\}) = 0 \quad j = 1,2,3
\]

(74)
Moreover, Lagrange's equations for the transverse displacements $v_i$ are

\[
\frac{1}{2} \left\{ \omega \right\}^T \left( \frac{\partial}{\partial v_i} \left[ \hat{J}_i(0) \right] \right) \{\omega\} + \{\omega\}^T \left( \frac{\partial}{\partial v_i} \left[ \hat{K}_i \right] \right) + \frac{1}{2} \Omega^2 \text{tr} \left( \frac{\partial}{\partial v_i} \left[ \hat{J}_i(0) \right] \right) \\
- \frac{3}{2} \Omega^2 \left\{ \varepsilon_a \right\}^T \left( \frac{\partial}{\partial v_i} \left[ \hat{J}_i(0) \right] \right) \{\varepsilon_a\} - \frac{3}{2} \left\{ \omega \right\}^T \frac{\partial}{\partial v_i} \left\{ \varepsilon \right\} + \rho_i \dot{v}_i \\
- \frac{\partial}{\partial x_i} \left( -P_{\times i} v_i + \frac{5}{2} EI_{zi} v_i' v_i'' \right) + \frac{3}{2} \frac{\partial^2}{\partial x_i^2} [-EI_{zi} v_i'' \right]
\]

\[
+ \frac{\partial}{\partial x_i} \left( \frac{\partial}{\partial v_i} \left[ \hat{J}_i(0) \right] \right) \{\omega\} + \text{tr} \left( \frac{\partial}{\partial v_i} \left[ \hat{J}_i(0) \right] \{\varepsilon_a\} \right) \\
- \frac{\partial}{\partial t} \left( \{\omega\}^T \frac{\partial}{\partial v_i} \left\{ \varepsilon \right\} + \rho_i \dot{v}_i \right) = 0 \quad \text{at } x_i = \varepsilon_i, \ i = 1, 2, \ldots, n \\
EI_{zi} v_i'' (1 - \frac{5}{2} v_i'^2) = 0
\]

\[
v_i = 0, v_i' = 0 \text{ at } x_i = 0, \ i = 1, 2, \ldots, n
\]  

which are subject to the boundary conditions.

Similarly, Lagrange's equations for the displacements $w_i$ are

\[
\frac{1}{2} \left\{ \omega \right\}^T \left( \frac{\partial}{\partial w_i} \left[ \hat{J}_i(0) \right] \right) \{\omega\} + \{\omega\}^T \left( \frac{\partial}{\partial w_i} \left[ \hat{K}_i \right] \right) + \frac{1}{2} \Omega^2 \text{tr} \left( \frac{\partial}{\partial w_i} \left[ \hat{J}_i(0) \right] \right)
\]
\[- \frac{3}{2} \omega^2 \{v_a\}^T \left[ \frac{\partial}{\partial w_i} [J_i^{(0)}] \right] \{v_a\} - \frac{3}{2} \frac{\partial}{\partial t} \left\{ \omega \right\} + \frac{3}{2} \frac{\partial}{\partial w_i} [K] + \rho_i \dot{w}_i \]

\[- \frac{3}{2} \frac{\partial}{\partial x_i} \left( -p_{xi} w_i' + \frac{5}{2} E_{yi} w_i w_i'^2 \right) + \frac{3^2}{2 \times 2} \left[ -E_{yi} w_i \right] (1 \right.

\[- \frac{5}{2} w_i'^2 \left) + p_{zi} = 0 , \ 0 < x_i < \varepsilon_i , \ i = 1, 2, ..., n \]

which are subject to the boundary conditions

\[- (p_{xi} - \frac{5}{2} E_{zi} w_i'^2) w_i' + \frac{3}{2} [E_{yi} w_i'^2(1 - \frac{5}{2} w_i'^2)]

\[+ \frac{1}{2} \omega^2 \left( \{\omega\}^T \frac{\partial}{\partial w_i} [J_i^{(0)}] \{\omega\} + \text{tr} \frac{\partial}{\partial w_i} [J_i^{(0)}] - 3 \{v_a\}^T \frac{\partial}{\partial w_i} [J_i^{(0)}] \{v_a\} \right)

\[- \frac{\partial}{\partial t} \left\{ \omega \right\} \times \frac{\partial}{\partial w_i} [K] + \rho_i \dot{W}_i \right\} = 0 \ \text{at} \ x_i = \varepsilon_i , \ i = 1, 2, ..., n \]

\[E_{yi} w_i'^2(1 - \frac{5}{2} w_i'^2) = 0 \]

\[w_i = 0, \ w_i' = 0 \ at \ x_i = 0 , \ i = 1, 2, ..., n \]

9. **Equilibrium Equations in Explicit Form**

For a gravity-gradient stabilized satellite, the angles \( \psi_j (j = 1, 2, 3) \) are measured relative to an orbiting system of axes. The orbit being circular, with the orbital angular velocity being equal to \( \Omega \), the orbital axes rotate relative to an inertial space with angular velocity \( \Omega \) about an axis normal to the orbital plane. This axis is denoted by \( c \) (see complete definition later). Hence, the angular velocity matrix \( \{\omega\} \) can be written as

36
\( \{\omega\} = \Omega\{l_c\} + \{\omega\}_r \) \hfill (77)

where \( \{l_c\} = \{l_c(\alpha_j)\} \) is the matrix of direction cosines between axis \( c \) and the reference system \( xyz \), and \( \{\omega\}_r = \{\omega(\alpha_j, \beta_j)\}_r \) is a matrix whose elements are the angular velocity components of system \( xyz \) relative to the orbital axes. They are linear combinations of the velocities \( \dot{\beta}_j \) \( (j = 1,2,3) \).

The equilibrium equations can be obtained by deleting from Eqs. (74) - (76) all the terms involving derivatives with respect to time. This implies that we can replace \( \{\omega\} \) by \( \Omega\{l_c\} \) in these equations. Hence, the nontrivial equilibrium must satisfy the general equations for the rotational motion

\[
\{l_c\}^T [J(\alpha)] \frac{\partial}{\partial \alpha_j} \{l_c\} - 3\{l_a\}^T [J(\alpha)] \frac{\partial}{\partial \alpha_j} \{l_a\} = 0, \quad j = 1,2,3
\] 

as well as the boundary-value problems defined by the differential equations

\[
\frac{1}{2} a^2 \left[ \{l_c\}^T \frac{\partial}{\partial v_i} [J_i(\alpha)] \{l_c\} + \text{tr} \left( \frac{\partial}{\partial v_i} [J_i(\alpha)] \right) - 3\{l_a\}^T \frac{\partial}{\partial v_i} [J_i(\alpha)] \{l_a\} \right]
\]

\[
+ \frac{\partial}{\partial x_i} \left( (p_{xi} - \frac{5}{2} E_{zi} v_i^2)v_i \right) - \frac{\partial^2}{\partial x_i^2} \left[ E_{zi} v_i^2(1 - \frac{5}{2} v_i^2) \right] = 0,
\]

\[0 < x_i < l_i, \quad i = 1,2,\ldots,n\] \hfill (79a)

and the boundary conditions

\[-(p_{xi} - \frac{5}{2} E_{zi} v_i^2)v_i + \frac{\partial}{\partial x_i} \left[ E_{zi} v_i^2(1 - \frac{5}{2} v_i^2) \right] + \frac{1}{2} \Omega^2 \left( \{l_c\}^T \frac{\partial}{\partial v_i} [J_i(\alpha)] \{l_c\} \right) \]
Moreover, it must satisfy a set of equations similar in structure to Eqs. (79), but with \( v_i \) replaced by \( w_i \).

10. The Variational Equations for the Discretized System

The variational equations for the discretized system were obtained earlier in the form (58), where the matrices \( [m] \), \( [g] \), and \( [k] \) are defined by Eqs. (49), (51), (53), and (57). Although the equations just mentioned have the advantage of revealing the symmetry of \( [m] \) and \( [k] \) and the skew-symmetry of \( [g] \), the formulas for deriving the elements of the matrices are not the most suitable from a computational point of view. Indeed, we wish to present a procedure whereby the actual derivation of the variational equations is performed by a digital computer.

Consistent with earlier notation, we shall denote quantities associated with equilibrium by the subscript 0 and perturbed quantities by the subscript 1. With this in mind, we can write the Lagrangian in the form

\[
L = L_0 + L_1
\]

\[
L_0 = \frac{1}{2} \{\omega\}_0^T [J^{(0)}]_0 \{\omega\}_0 + \{\omega\}_0^T [K]_0 + T_{EO} + \frac{1}{2} \Omega^2 \operatorname{tr}[J^{(0)}]_0
\]

\[
- \frac{3}{2} \Omega^2 \{z_a\}_0^T [J^{(0)}]_0 \{z_a\}_0 - V_{EO}
\]

(80)

(81)
and

\[ L_1 = \{\omega\}^T \{J(0)\}_0 \{\omega\} + \frac{1}{2} \{\omega\}^T \{J(0)\}_1 \{\omega\} + \frac{1}{2} \{\omega\}^T \{J(0)\}_1 \{\omega\} \]

\[ + \{\omega\}^T \{J(0)\}_1 \{\omega\}_0 + \{\omega\}^T \{K\}_0 + \{\omega\}^T \{K\}_1 + \{\omega\}_0 \{K\}_1 + T_{E1} \]

\[ + \frac{1}{2} \Omega^2 \text{tr}\{J(0)\}_1 - 3\Omega^2 \{\varepsilon_a\}_0 \{J(0)\}_0 \{\varepsilon_a\}_0 - \frac{3}{2} \Omega^2 \{\varepsilon_a\}_1 \{J(0)\}_0 \{\varepsilon_a\}_1 \]

\[ - \frac{3}{2} \Omega^2 \{\varepsilon_a\}_0 \{J(0)\}_1 \{\varepsilon_a\}_0 - 3\Omega^2 \{\varepsilon_a\}_1 \{J(0)\}_1 \{\varepsilon_a\}_0 - V_{E1} \]  

\[ (82) \]

in which

\[ \{\omega\} = \{\omega\}_0 + \{\omega\}_1 = \Omega \{\varepsilon_c\}_0 + \frac{3}{\Sigma} \sum_{i=1} \left[ \frac{\partial}{\partial \varepsilon_i} \{\omega\} \right] q_i + \frac{3}{\Sigma} \sum_{i=1} \left[ \frac{\partial}{\partial \varepsilon_i} \{\omega\} \right] \dot{q}_i \]

\[ + \frac{1}{2} \frac{3}{\Sigma} \sum_{i=1} \sum_{j=1} \left[ \frac{\partial^2}{\partial \varepsilon_i \partial \varepsilon_j} \{\omega\} \right] q_i q_j + 2 \frac{3}{\Sigma} \sum_{i=1} \left[ \frac{\partial^2}{\partial \varepsilon_i \partial \varepsilon_j} \{\omega\} \right] q_i \dot{q}_j \]  

\[ (83) \]

\[ \{\varepsilon_a\} = \{\varepsilon_a\}_0 + \{\varepsilon_a\}_1 = \{\varepsilon_a\}_0 + \frac{3}{\Sigma} \sum_{i=1} \left[ \frac{\partial}{\partial \varepsilon_i} \{\varepsilon_a\} \right] q_i \]

\[ + \frac{1}{2} \frac{3}{\Sigma} \sum_{i=1} \sum_{j=1} \left[ \frac{\partial^2}{\partial \varepsilon_i \partial \varepsilon_j} \{\varepsilon_a\} \right] q_i q_j \]  

\[ (84) \]

\[ [J(0)] = \sum_{i=0}^{n} [\varepsilon_i] \{J_i\} [\varepsilon_i] = \sum_{i=0}^{n} [\varepsilon_i] \left[ \int_0^{\varepsilon_i} \left[ \frac{\partial \varepsilon_i}{\partial \varepsilon_i} \right] dx_i + [J_i(\varepsilon_i)] \right] [\varepsilon_i] \]

\[ = [J(0)]_0 + [J(0)]_1 = [J(0)]_0 + \sum_{i=0}^{n} [\varepsilon_i] \left[ \int_0^{\varepsilon_i} \left[ \frac{\partial \varepsilon_i}{\partial v_{i0}} \right] v_{i1} \right] \]
\[
\begin{align*}
+ & \left[ \frac{a^2 J_i}{aw_0} \right] w_{11} + \frac{1}{2} \left[ \frac{a^2 J_i}{av_0^2} \right] v_{i1}^2 + \left[ \frac{a^2 J_i}{aw_0^2} \right] v_{i1} w_{11} + \frac{1}{2} \left[ \frac{a^2 J_i}{aw_0^2} \right] w_{11}^2 \ dx_i \\
+ & \left[ \frac{a J_i(e_i)}{av_0} \right] v_{i1}(e_i) + \left[ \frac{a J_i(e_i)}{aw_0} \right] w_{11}(e_i) + \frac{1}{2} \left[ \frac{a^2 J_i(e_i)}{aw_0^2} \right] v_{i1}^2(e_i) \\
+ & \left[ \frac{a^2 J_i(e_i)}{av_0^2 aw_0} \right] v_{i1}(e_i) w_{11}(e_i) + \frac{1}{2} \left[ \frac{a^2 J_i(e_i)}{aw_0^2} \right] w_{11}^2(e_i) \ [e_i] \quad (85)
\end{align*}
\]

\[
\{K\} = \sum_{i=1}^{n} \left[ \int_{0}^{e_i} \left\{ \hat{k}_i \right\} dx_i + \{K_i(e_i)\} \right]
\]

\[
\{K\}_0 = 0
\]

\[
\{K\}_1 = \sum_{i=1}^{n} \int_{0}^{e_i} \left[ \left\{ \frac{a K_i}{av_0} \right\} v_{i1} + \left\{ \frac{a K_i}{aw_0} \right\} w_{11} + \left\{ \frac{a K_i}{aw_0^2} \right\} v_{i1}^2 + \left\{ \frac{a K_i}{aw_0^2} \right\} w_{11}^2 \right. \\
+ \left\{ \frac{a^2 K_i}{av_0^2 aw_0} \right\} v_{i1} w_{11} + \left\{ \frac{a^2 K_i}{av_0^2 aw_0} \right\} v_{i1}^2 w_{11} + \left\{ \frac{a K_i(e_i)}{av_0} \right\} v_{i1}(e_i) \\
+ \left\{ \frac{a K_i(e_i)}{aw_0} \right\} w_{11}(e_i) + \left\{ \frac{a K_i(e_i)}{aw_0^2} \right\} v_{i1}(e_i)^2 + \left\{ \frac{a K_i(e_i)}{aw_0^2} \right\} w_{11}(e_i)^2 \right] \ dx_i + \left. \left\{ \frac{a K_i(e_i)}{av_0} \right\} v_{i1}(e_i) \\
+ \left\{ \frac{a K_i(e_i)}{aw_0} \right\} w_{11}(e_i) + \left\{ \frac{a^2 K_i(e_i)}{av_0^2 aw_0} \right\} v_{i1}(e_i) w_{11}(e_i) + \left\{ \frac{a^2 K_i(e_i)}{av_0^2 aw_0} \right\} v_{i1}(e_i)^2 w_{11}(e_i) \right) \quad (86)
\]

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Note that Eqs. (83) - (86) represent Taylor's series expansions of \( \{ \omega \} \), \( \{ \alpha_a \} \), \( [J(0)] \), and \( \{ K \} \) about equilibrium. Moreover, we have

\[
T_{E0} = 0 \tag{87}
\]

and

\[
T_{E1} = \frac{1}{2} \sum_{i=1}^{n} \left[ \int_{0}^{L_i} \rho_i \left( \dot{u}_i \right) \left( \dot{u}_i \right) dx_i + m_i \left( \dot{u}_i \right) \left( \dot{u}_i \right) \bigg|_{x_i = i} \right] \\
= \frac{1}{2} \sum_{i=1}^{n} \int_{0}^{L_i} \rho_i \left( \dot{v}_i \right) \left( \dot{v}_i \right) dx_i + m_i \left( \dot{v}_i \right) \left( \dot{v}_i \right) \bigg|_{x_i = i} \tag{88}
\]

as well as

\[
V_{E0} = \frac{1}{2} \sum_{i=1}^{n} \int_{0}^{L_i} \left[ P_i \left( \frac{\partial v_{i0}}{\partial x_i} \right)^2 + \left( \frac{\partial w_{i0}}{\partial x_i} \right)^2 \right] + E I_{zi} \left( \frac{\partial^2 v_{i0}}{\partial x_i^2} \right) \left[ 1 - \frac{5}{2} \left( \frac{\partial v_{i0}}{\partial x_i} \right)^2 \right] \\
\quad + E I_{yi} \left( \frac{\partial^2 w_{i0}}{\partial x_i^2} \right) \left[ 1 - \frac{5}{2} \left( \frac{\partial w_{i0}}{\partial x_i} \right)^2 \right] dx_i \tag{89}
\]

and

\[
V_{E1} = \frac{1}{2} \sum_{i=1}^{n} \int_{0}^{L_i} \left[ P_i \left( \frac{\partial v_{i1}}{\partial x_i} \right) \left( \frac{\partial v_{i1}}{\partial x_i} \right) + \frac{\partial w_{i1}}{\partial x_i} \left( \frac{\partial w_{i1}}{\partial x_i} \right) + \left( \frac{\partial v_{i1}}{\partial x_i} \right)^2 + \left( \frac{\partial w_{i1}}{\partial x_i} \right)^2 \right] \\
\quad - 5 E I_{zi} \left( \frac{\partial v_{i0}}{\partial x_i} \right) \left( \frac{\partial^2 v_{i0}}{\partial x_i^2} \right) \left( \frac{\partial v_{i1}}{\partial x_i} \right) - \frac{5}{2} E I_{zi} \left( \frac{\partial^2 v_{i0}}{\partial x_i^2} \right) \left( \frac{\partial v_{i1}}{\partial x_i} \right)^2 \\
\quad + 2 E I_{zi} \left[ 1 - \frac{5}{2} \left( \frac{\partial v_{i0}}{\partial x_i} \right)^2 \right] \left( \frac{\partial^2 v_{i0}}{\partial x_i^2} \right) \left( \frac{\partial v_{i1}}{\partial x_i} \right)^2 + E I_{zi} \left[ 1 - \frac{5}{2} \left( \frac{\partial v_{i0}}{\partial x_i} \right)^2 \right] \left( \frac{\partial^2 v_{i1}}{\partial x_i^2} \right)^2
\]

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To obtain the variational equations in terms of the discrete co-
ordinates \( q_j(t) \) (\( j = 1, 2, \ldots, 2np+3 \)), we must insert the modal expansions (47) into \( L_1 \) and perform the indicated integrations over the spatial
variables \( x_i \) (\( i = 1, 2, \ldots, n \)). Because the resulting expressions are very
lengthy, we shall not write them explicitly, but proceed with the derivation
of Lagrange's equations instead. To this end, it will prove convenient to
denote constant terms by the subscript \( c \) and terms that are linear in the
generalized coordinates \( q_j(t) \) and generalized velocities \( \dot{q}_j(t) \) by the sub-
script \( i \). This enables us to write

\[
\frac{\partial L_1}{\partial q_j} \approx \left( \frac{\partial}{\partial q_j} \{\omega\}^T \right)_i \left( [J(0)]_0 \{\omega\}_0 \right) + \left( \frac{\partial}{\partial q_j} \{\omega\}^T \right)_c \left( [J(0)]_c \{\omega\}_c \right) + \left( [J(0)]_i \{\omega\}_0 + (K)_i \right) \quad j = 1, 2, 3
\]

(91a)
\[
\frac{\partial L_1}{\partial q_j} = \left\{ \omega \right\}_T \left[ \frac{\partial}{\partial q_j} \left\{ K \right\}_c + \left\{ \omega \right\}_0 \right] + \left\{ \omega \right\}_0 \left[ \frac{\partial}{\partial q_j} \left\{ K \right\}_c \right] + \left\{ \frac{3}{3q_j} TEI \right\}_c
\]
\]
\[j = 4,5,\ldots,2np+3 \quad (91b)\]

\[
\frac{\partial L_1}{\partial q_j} = \left( \frac{\partial}{\partial q_j} \left\{ \omega \right\}_T \right)_c \left[ \left\{ J(0) \right\}_0 \left\{ \omega \right\}_0 + \left\{ \left( K \right\}_c \right\}_c \right] - 3 \Omega^2 \left[ \frac{\partial}{\partial q_j} \left\{ \epsilon a \right\} T \right] \left[ \left\{ J(0) \right\}_0 \left\{ \epsilon a \right\}_0 \right] - 3 \Omega^2 \left[ \frac{\partial}{\partial q_j} \left\{ \epsilon a \right\} T \right] \left[ \left\{ J(0) \right\}_0 \left\{ \epsilon a \right\}_0 \right]
\]
\[j = 1,2,3 \quad (91c)\]

\[
\frac{\partial L_1}{\partial q_j} = \left\{ \omega \right\}_0 \left[ \frac{1}{2} \left( \frac{\partial}{\partial q_j} \left\{ J(0) \right\}_0 \right)_c \left\{ \omega \right\}_0 + \left( \frac{\partial}{\partial q_j} \left\{ K \right\}_c \right) \right] + \left\{ \left( K \right\}_c \right\}_c \left\{ \omega \right\}_0 + \left( \frac{\partial}{\partial q_j} \left\{ K \right\}_c \right) \]
\[+ \frac{1}{2} \Omega^2 \left( \frac{\partial}{\partial q_j} \left\{ J(0) \right\}_0 \right)_c - \frac{3}{2} \Omega^2 \left( \frac{\partial}{\partial q_j} \left\{ J(0) \right\}_0 \right)_c \left( \epsilon a \right)_0 - 3 \Omega^2 \left[ \frac{\partial}{\partial q_j} \left\{ \epsilon a \right\} T \right] \left[ \left\{ J(0) \right\}_0 \left\{ \epsilon a \right\}_0 \right] + \left( \frac{\partial}{\partial q_j} \left\{ V E1 \right\}_c \right)
\]
\[j = 4,5,\ldots,2np+3 \quad (91d)\]

which enables us to write Lagrange's equations for the perturbed motion in the compact form.
\[
\left( \frac{a}{\partial q_j} \right)_{T} \left( [J(0)]_0 \{\omega \}_{0} + \left[ \frac{a}{\partial q_j} \right]_{T} \left[ [J(0)]_0 \{\omega \}_{0} \right] \right)
\]

\[
+ \left[ [J(0)]_1 \{\omega \}_{0} + \left[ \{K\}_{1} \right]_{c} \right) - 3 \Omega^2 \left[ \frac{a}{\partial q_j} \left[ \{z_a \}_{1} \right]_{c} \left[ [J(0)]_0 \{\varepsilon_a \}_{0} \right] \right]
\]

\[
- 3 \Omega^2 \left[ \frac{a}{\partial q_j} \left[ \{z_a \}_{1} \right]_{c} \left[ [J(0)]_0 \{\varepsilon_a \}_{0} \right] \right]
\]

\[
+ \frac{d}{dt} \left( \left[ \frac{a}{\partial q_j} \left[ \{\omega \}_{1} \right]_{c} \left[ [J(0)]_0 \{\omega \}_{0} \right] + \left[ \frac{a}{\partial q_j} \left[ \{\omega \}_{1} \right]_{c} \left[ [J(0)]_0 \{\omega \}_{0} \right] \right) = 0 \right), j = 1, 2, 3
\]

\[
\left\{ \omega \right\}_{0} \left( \frac{1}{2} \left[ \frac{a}{\partial q_j} \left[ [J(0)]_1 \right] \right]_{c} \left[ \{\omega \}_{0} + \left[ \frac{a}{\partial q_j} \{K\}_{1} \right]_{c} \right)
\]

\[
+ \left[ \{\omega \}_{1} \right]_{c} \left( \left[ \frac{a}{\partial q_j} \left[ [J(0)]_1 \right] \right]_{c} \left[ \{\omega \}_{0} + \left[ \frac{a}{\partial q_j} \{K\}_{1} \right] \right) \right.
\]

\[
+ \frac{1}{2} \Omega^2 \frac{a}{\partial q_j} \left[ \text{tr} [J(0)]_1 \right]_{c} - \frac{3}{2} \Omega^2 \left[ \varepsilon_a \right]_0 \left[ [J(0)]_1 \right]_{c} \left[ \varepsilon_a \right]_0
\]

\[
- 3 \Omega^2 \left[ \{\varepsilon_a \}_{1} \right]_{c} \left[ [J(0)]_1 \right]_{c} \left[ \varepsilon_a \right]_0 + \left[ \frac{a}{\partial q_j} \left[ V_{E1} \right] \right]_{c}
\]

\[
- \frac{d}{dt} \left( \left[ \{\omega \}_{1} \right]_{c} \left[ \frac{a}{\partial q_j} \left[ \{K\}_{1} \right] \right] + \left[ \omega \right]_0 \left[ [J(0)]_1 \right]_{c} + \left[ \frac{a}{\partial q_j} \left[ T_{E1} \right] \right]_{c} \right) = 0
\]

\[
j = 4, 5, \ldots, 2np+3
\]
As mentioned already, the advantage of Lagrange's equations (92) over those derived in Sec. 4 is that Eqs. (92) permit automatic derivation by means of a digital computer.

Before specializing the equations to a particular satellite, let us derive an expression for the axial force $P_{x_i}$ in terms of matrix notation. The axial force $P_{x_i}$ is due to centrifugal and differential gravity effects. Introducing the modified potential energy density associated with member $i$

$$\dot{V}_{mod} = -\frac{1}{2} \omega^2 (\{\xi_c\}^T [\dot{J}_i^{(0)}] \{\xi_c\} + \text{tr}[\dot{J}_i^{(0)}]) - 3 \{\xi_a\}^T [\dot{J}_i^{(0)}] \{\xi_a\}$$

- $\frac{1}{2} \omega^2 (\{\xi_c\}^T [J_i^{(0)}(x_i)] \{\xi_c\} + \text{tr}[J_i^{(0)}(x_i)])$

- $3 \{\xi_a\}^T [J_i^{(0)}(x_i)] \{\xi_a\} \delta(x_i - \xi_i)$  \hspace{1cm} (93)

where the terms inside parentheses and multiplying $\delta(x_i - \xi_i)$ are due to the tip masses, the axial force density can be written in the form

$$P_{x_i}(x_i) = \frac{\partial \dot{V}_{mod}}{\partial x_i} = \frac{3 \dot{V}_{mod}}{\partial x_i}$$

= $\frac{1}{2} \omega^2 (\{\xi_c\}^T [\dot{J}_i^{(0)}]' \{\xi_c\} + \text{tr}[\dot{J}_i^{(0)}]') - 3 \{\xi_a\}^T [\dot{J}_i^{(0)}]' \{\xi_a\}$  \hspace{1cm} (94)

in which we introduced the notation

$$[\dot{J}_i^{(0)}]' = \frac{3}{\partial x_i} [\dot{J}_i^{(0)}] + \frac{3}{\partial x_i} [\dot{J}_i^{(0)}(x_i)] \delta(x_i - \xi_i)$$  \hspace{1cm} (95)

Observing from Eq. (67) that $P_{x_i}$ is multiplied by $(v_i^1 + w_i^2)$, we ignore any transverse terms in $[\dot{J}_i^{(0)}]'$, so that using the first of Eqs. (68) and Eqs. (69) we obtain the approximation
Inserting Eqs. (95) and (96) into Eq. (94), we can write the axial force \( P_{x_i} \) at any point \( x_i \) in the form of the integral

\[
P_{x_i} = \int_{x_i}^{\xi} \hat{p}_{\xi_i}(\xi_i) d\xi_i = \omega^2 \{\xi_0\}^T \left[ \int_{x_i}^{\xi} \hat{J}_i^{(0)}(\xi_i) d\xi_i \right] \{\xi_0\}
\]

\[+ \text{tr}\left( \int_{x_i}^{\xi} \hat{J}_i^{(0)}(\xi_i) d\xi_i \right) - 3\{\xi_0\}^T \left[ \int_{x_i}^{\xi} \hat{J}_i^{(0)}(\xi_i) d\xi_i \right] \{\xi_0\} \]

where, assuming that \( \rho_i = \text{const} \), we have

\[
\left[ \int_{x_i}^{\xi} \hat{J}_i^{(0)}(\xi_i) d\xi_i \right] = \langle \rho_i [(h_{x_i} + \varepsilon_i)^2 - (h_{x_i} + x_i)^2] \rangle
\]

\[+ \frac{1}{2} \rho_i [(h_{x_i} + \varepsilon_i)^2 - (h_{x_i} + x_i)^2] \]

\[+ 2m_i (h_{x_i} + \varepsilon_i) \langle \varepsilon_1 \rangle \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 1 \\ 0 & 1 & 0 \end{bmatrix} \{\varepsilon_1\} \]

It follows that the desired expression has the form

\[
P_{x_i} = \omega^2 \left( \frac{1}{2} \rho_i [(h_{x_i} + \varepsilon_i)^2 - (h_{x_i} + x_i)^2] \right)
\]

\[+ m_i (h_{x_i} + \varepsilon_i) \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 1 \\ 0 & 1 & 0 \end{bmatrix} \{\varepsilon_1\} - \text{tr}[\{\varepsilon_1\}]^T \begin{bmatrix} 0 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \{\varepsilon_1\} \]

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a. Equations of motion

Next let us specialize the equations to the case of a satellite consisting of a rigid core with six flexible booms, as shown in Fig. 6. First, we wish to determine the matrices $[\ell_i]$ of the direction cosines between axes $x_i$,$y_i$,$z_i$ and $xyz$. From Fig. 6, it is easy to verify that

$$[\ell_1] = \begin{bmatrix} c\alpha & s\alpha & 0 \\ -s\alpha & c\alpha & 0 \\ 0 & 0 & 1 \end{bmatrix}, \quad [\ell_2] = \begin{bmatrix} -c\alpha & s\alpha & 0 \\ -s\alpha & -c\alpha & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

$$[\ell_3] = \begin{bmatrix} -c\alpha & -s\alpha & 0 \\ s\alpha & -c\alpha & 0 \\ 0 & 0 & 1 \end{bmatrix}, \quad [\ell_4] = \begin{bmatrix} c\alpha & -s\alpha & 0 \\ s\alpha & c\alpha & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

$$[\ell_5] = \begin{bmatrix} 0 & s\beta & c\beta \\ 1 & 0 & 0 \\ 0 & c\beta & -s\beta \end{bmatrix}, \quad [\ell_6] = \begin{bmatrix} 0 & -s\beta & -c\beta \\ 1 & 0 & 0 \\ 0 & -c\beta & s\beta \end{bmatrix}$$

where $s\alpha = \sin \alpha$, $c\alpha = \cos \alpha$, $s\beta = \sin \beta$, and $c\beta = \cos \beta$. Moreover, to write the angular velocity matrix $\{\omega\}$ in explicit form, we must specify the rotations $\theta_j$ ($j = 1,2,3$). Assuming that system xyz is obtained from system abc by means of the rotations $\theta_2$ about y, $-\theta_1$ about x, and $\theta_3$ about z, and recalling that axes abc rotate about c with the constant angular velocity $\omega$, matrix $\{\omega\}$ can be shown to have the expression
\[
(\omega) = \Omega \left( \begin{array}{ccc}
-s_3 & c_3 & 0 \\
-s_2 s_3 & -s_2 c_2 & s_2 c_3 \\
0 & s_2 & 1
\end{array} \right) + \left( \begin{array}{ccc}
-c_3 & s_3 & 0 \\
s_3 & c_3 & 0 \\
0 & s_3 & 1
\end{array} \right) (\dot{\theta}_1) \]
(101)

where \( s_1 = \sin \theta_1, c_1 = \cos \theta_1 \), etc. Because the direction of the radius vector \( R_c \) coincides with that of axis \( a \) at all times, the direction matrix \( \{l_a\} \) can be written as

\[
(\{l_a\}) = \left( \begin{array}{ccc}
-c_2 c_3 - s_2 s_1 c_2 c_3 \\
-c_2 s_3 + s_2 s_1 c_2 c_3 \\
c_2 s_2
\end{array} \right)
\]
(102)

It will prove convenient to rewrite matrices \( \{(\omega) \} \) and \( \{l_a\} \) as follows

\[
(\omega) = \left[ \theta \right]_3 \left[ \theta^* \right]_1 (\dot{\theta}) + \{\dot{\theta}_3\} + \Omega \left[ \theta \right]_3 \left[ \theta^* \right]_1 (\dot{\theta})_2
\]
(103)

where

\[
[\theta]_3 = \begin{bmatrix}
c_3 & s_3 & 0 \\
-s_3 & c_3 & 0 \\
0 & 0 & 1
\end{bmatrix}, \quad [\theta^*]_1 = \begin{bmatrix}
-1 & 0 & 0 \\
0 & c_1 & 0 \\
0 & s_1 & 0
\end{bmatrix}, \quad \{\dot{\theta}_3\} = \begin{bmatrix}
0 \\
0 \\
\dot{\theta}_3
\end{bmatrix}
\]
(104)

\[
[\theta]_1 = \begin{bmatrix}
1 & 0 & 0 \\
0 & c_1 & -s_1 \\
0 & s_1 & c_1
\end{bmatrix}, \quad \{\theta\}_2 = \begin{bmatrix}
-s_2 \\
0 \\
c_2
\end{bmatrix}, \quad \{\dot{\theta}\}_2 = \begin{bmatrix}
0 \\
0 \\
0
\end{bmatrix}
\]

Introducing Eq. (103) into (4), and recalling Eq. (68), the kinetic energy becomes

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\[ T = \frac{1}{2} \{\omega\}^T \begin{bmatrix} J^{(O)} \end{bmatrix} \{\omega\} + \{\omega\}^T \{K\} + T_E \]

\[ = \frac{1}{2} \{\dot{\theta}\}^T \begin{bmatrix} e^* & T \end{bmatrix} \begin{bmatrix} e \end{bmatrix}_3 \begin{bmatrix} J^{(O)} \end{bmatrix} \begin{bmatrix} e \end{bmatrix}_3 \begin{bmatrix} e^* \end{bmatrix}_1 \{\dot{\theta}\} + \{\dot{\theta}\}^T \begin{bmatrix} e^* \end{bmatrix}_3 \begin{bmatrix} J^{(O)} \end{bmatrix} \begin{bmatrix} e \end{bmatrix}_3 \begin{bmatrix} e \end{bmatrix}_3 \begin{bmatrix} e^* \end{bmatrix}_1 \{\dot{\theta}\}_3 \]

\[ + \frac{1}{2} \{\dot{\theta}\}^T \begin{bmatrix} e \end{bmatrix}_3 \begin{bmatrix} J^{(O)} \end{bmatrix} \{\dot{\theta}\}_3 + \{\dot{\theta}\}^T \begin{bmatrix} e^* \end{bmatrix}_1 \begin{bmatrix} e \end{bmatrix}_3 \begin{bmatrix} J^{(O)} \end{bmatrix} \begin{bmatrix} e \end{bmatrix}_3 \begin{bmatrix} e^* \end{bmatrix}_1 \{\dot{\theta}\}_3 \]

\[ + \alpha \{\dot{\theta}\}^T \begin{bmatrix} e \end{bmatrix}_2 \begin{bmatrix} e \end{bmatrix}_3 \begin{bmatrix} J^{(O)} \end{bmatrix} \begin{bmatrix} e \end{bmatrix}_3 \begin{bmatrix} e^* \end{bmatrix}_1 \{\dot{\theta}\}_2 + \frac{1}{2} \Omega^2 \{\dot{\theta}\}^T \begin{bmatrix} e \end{bmatrix}_2 \begin{bmatrix} e \end{bmatrix}_3 \begin{bmatrix} J^{(O)} \end{bmatrix} \begin{bmatrix} e \end{bmatrix}_3 \begin{bmatrix} e \end{bmatrix}_3 \begin{bmatrix} e \end{bmatrix}_1 \{\dot{\theta}\}_2 \]

\[ = \frac{1}{2} \{\dot{\theta}\}^T \begin{bmatrix} e^* \end{bmatrix}_1 \begin{bmatrix} J^* \end{bmatrix} \begin{bmatrix} e^* \end{bmatrix}_1 \{\dot{\theta}\} + \{\dot{\theta}\}^T \begin{bmatrix} e^* \end{bmatrix}_1 \begin{bmatrix} J \end{bmatrix} \begin{bmatrix} e \end{bmatrix}_3 \{\dot{\theta}\}_3 \]

\[ + \{\dot{\theta}\}^T \begin{bmatrix} J^* \end{bmatrix}_1 \{K^*\} + \frac{1}{2} \Omega^2 \{\dot{\theta}\}^T \begin{bmatrix} J^* \end{bmatrix}_1 \{K\}_3 + \{\dot{\theta}\}^T \begin{bmatrix} J^* \end{bmatrix}_1 \{K\}_3 \{\dot{\theta}\}_3 \]

\[ + \alpha \{\dot{\theta}\}^T \begin{bmatrix} J^* \end{bmatrix}_1 \{K\}_3 + \frac{1}{2} \Omega^2 \{\dot{\theta}\}^T \begin{bmatrix} J^* \end{bmatrix}_1 \{\dot{\theta}\}_2 \{\dot{\theta}\}_2 \]

\[ = \frac{1}{2} \{\dot{\theta}\}^T \begin{bmatrix} e^* \end{bmatrix}_1 \begin{bmatrix} J^* \end{bmatrix} \begin{bmatrix} e^* \end{bmatrix}_1 \{\dot{\theta}\}_1 \{\dot{\theta}\}_1 \]

\[ + \{\dot{\theta}\}^T \begin{bmatrix} J^* \end{bmatrix}_1 \{K^*\} + \frac{1}{2} \Omega^2 \{\dot{\theta}\}^T \begin{bmatrix} J^* \end{bmatrix}_1 \{K\}_1 \{\dot{\theta}\}_1 \{\dot{\theta}\}_2 \]

\[ = \frac{1}{2} \{\dot{\theta}\}^T \begin{bmatrix} e^* \end{bmatrix}_1 \begin{bmatrix} J^* \end{bmatrix} \begin{bmatrix} e^* \end{bmatrix}_1 \{\dot{\theta}\}_1 \{\dot{\theta}\}_1 \]

\[ + \{\dot{\theta}\}^T \begin{bmatrix} J^* \end{bmatrix}_1 \{K^*\} + \frac{1}{2} \Omega^2 \{\dot{\theta}\}^T \begin{bmatrix} J^* \end{bmatrix}_1 \{K\}_1 \{\dot{\theta}\}_1 \{\dot{\theta}\}_2 \]

\[ = \frac{1}{2} \Omega^2 \{\dot{\theta}\}^T \begin{bmatrix} J^* \end{bmatrix} \begin{bmatrix} e^* \end{bmatrix}_1 \{\dot{\theta}\}_1 \{\dot{\theta}\}_1 \]

\[ + \{\dot{\theta}\}^T \begin{bmatrix} J^* \end{bmatrix}_1 \{K^*\} + \frac{1}{2} \Omega^2 \{\dot{\theta}\}^T \begin{bmatrix} J^* \end{bmatrix}_1 \{K\}_1 \{\dot{\theta}\}_1 \{\dot{\theta}\}_2 \]

\[ = \frac{1}{2} \Omega^2 \{\dot{\theta}\}^T \begin{bmatrix} J^* \end{bmatrix} \begin{bmatrix} e^* \end{bmatrix}_1 \{\dot{\theta}\}_1 \{\dot{\theta}\}_1 \]

\[ + \{\dot{\theta}\}^T \begin{bmatrix} J^* \end{bmatrix}_1 \{K^*\} + \frac{1}{2} \Omega^2 \{\dot{\theta}\}^T \begin{bmatrix} J^* \end{bmatrix}_1 \{K\}_1 \{\dot{\theta}\}_1 \{\dot{\theta}\}_2 \]

where \( [J^*] = [e]_3 \begin{bmatrix} J^{(O)} \end{bmatrix} \begin{bmatrix} e \end{bmatrix}_3 \) and \( \{K^*\} = [e]_3 \{K\} \). Moreover, inserting Eq. (103) into (5), and recognizing that \( \{\zeta_a\}^T = -\{\{e\}_2\}^T \{e\}_3 \{e\}_3 \), we obtain the gravitational potential energy in the form

\[ V_G = -\frac{1}{2} \Omega^2 \text{tr} \begin{bmatrix} J^{(O)} \end{bmatrix} + \frac{3}{2} \Omega^2 \{\{e\}_2\}^T \begin{bmatrix} e \end{bmatrix}_3 \begin{bmatrix} J^{(O)} \end{bmatrix} \begin{bmatrix} e \end{bmatrix}_3 \begin{bmatrix} e \end{bmatrix}_1 \{\{e\}_2\}_1 \]

\[ = -\frac{1}{2} \Omega^2 \text{tr} \begin{bmatrix} J^{(O)} \end{bmatrix} + \frac{3}{2} \Omega^2 \{\{e\}_2\}^T \begin{bmatrix} J^* \end{bmatrix}_1 \begin{bmatrix} e \end{bmatrix}_3 \begin{bmatrix} e \end{bmatrix}_3 \{\{e\}_2\}_1 \]

where primes indicate differentiation with respect to \( e_2 \). Expanding the matrix involved in \( T \) and \( V_G \), and recalling that \( L = T - V_G - V_{EA} - V_{EB} \), the Lagrangian \( L \) can be written in the form

\[ L = \frac{1}{2} \begin{bmatrix} J^* \end{bmatrix}_1 \{\dot{\theta}\}_1^2 + \begin{bmatrix} J^* \end{bmatrix}_{22} c^2 \dot{\theta}_1 + \begin{bmatrix} J^* \end{bmatrix}_{33} s^2 \dot{\theta}_1 + 2 \begin{bmatrix} J^* \end{bmatrix}_{23} s \dot{\theta}_1 c \dot{\theta}_1 \]

\[ \dot{\theta}_2^2 \]

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To obtain Lagrange’s equations for the rotational motion, we introduce Eq. (107) into Eqs. (35), and obtain

\[
[(J_{33} - J_{22}) s_{\theta_1} c_{\theta_1} + J_{23} (c_{\theta_2} - s_{\theta_2})] \ddot{\theta}_2 + (J_{12} s_{\theta_1} - J_{13} c_{\theta_1}) \dot{\theta}_1 \dot{\theta}_2
\]
\[
\begin{align*}
& \quad + (-J_{33}^* s_0 \theta_1 + J_{33}^* c_0 \theta_1) \dot{\theta}_2 \dot{\theta}_3 - (K_2^* s_0 \theta_1 - K_3^* c_0 \theta_1) \dot{\theta}_2 \\
& \quad + \Omega \left( J_{12}^* c_0 \theta_1 c_0 + J_{13}^* s_0 \theta_1 c_0 \right) \dot{\theta}_1 + \left[ J_{12}^* s_0 \theta_1 s_0 - J_{13}^* c_0 \theta_1 s_0 \right] \dot{\theta}_2 \\
& \quad + (J_{33}^* - J_{22}^*)(c^2 \theta_1 - s^2 \theta_1) c_0 - 4 J_{23}^* c_0 s_0 \theta_1 c_0 \dot{\theta}_3 \\
& \quad - (J_{33}^* s_0 \theta_1 c_0 + J_{23}^* c_0 \theta_1 c_0) \dot{\theta}_3 - K_2^* c_0 \theta_1 c_0 - K_3^* s_0 \theta_1 c_0 \\
& \quad + \Omega^2 \left[ 4 J_{12}^* s_0 \theta_1 s_0 c_0 + (J_{22}^* - J_{33}^*) s_0 \theta_1 c_0 \left( c^2 \theta_2 - 3s^2 \theta_2 \right) \\
& \quad + 4 J_{13}^* s_0 \theta_1 s_0 c_0 + J_{23}^* c_0 \theta_1 - s_0 \theta_1 \left( 3s^2 \theta_2 - c^2 \theta_2 \right) \right] \\
& \quad - \frac{d}{dt} \left[ J_{11}^* \dot{\theta}_1 - (J_{12}^* c_0 \theta_1 + J_{13}^* s_0 \theta_1) \dot{\theta}_2 - J_{13}^* \dot{\theta}_3 - K_1^* \\
& \quad + \Omega \left( J_{11}^* \dot{\theta}_2 + J_{12}^* s_0 \theta_1 c_0 - J_{13}^* c_0 \theta_1 c_0 \right) \right] = 0 \\
& \quad \Omega \left[ J_{11}^* c_0 - J_{12}^* s_0 \theta_1 c_0 + J_{13}^* s_0 \theta_1 c_0 \right] \dot{\theta}_1 + \left[ -J_{12}^* c_0 \theta_1 c_0 + J_{22}^* s_0 \theta_1 s_0 c_0 \right] \\
& \quad - J_{33}^* c_0 s_0 \theta_1 s_0 c_0 - J_{13}^* s_0 \theta_1 c_0 - J_{23}^* s_0 \theta_1 c_0 \left( c^2 \theta_1 - s^2 \theta_1 \right) \dot{\theta}_2 - (J_{33}^* c_0 \theta_1 s_0 \theta_1 \\
& \quad + J_{13}^* c_0 - J_{23}^* s_0 \theta_1 c_0) \dot{\theta}_3 - K_1^* c_0 \theta_1 + K_2^* s_0 \theta_1 s_0 - K_3^* c_0 \theta_1 s_0 \theta_1 \\
& \quad + \Omega^2 \left[ 4 J_{11}^* s_0 \theta_2 c_0 + 4 J_{12}^* s_0 \theta_1 \left( c^2 \theta_2 - s^2 \theta_2 \right) - 4 J_{22}^* s^2 \theta_1 s_0 c_0 \right] \\
& \quad - 4 J_{13}^* c_0 \theta_1 \left( c^2 \theta_2 - s^2 \theta_2 \right) + 8 J_{23}^* s_0 \theta_1 s_0 c_0 \theta_1 c_0 - 4 J_{33}^* c^2 \theta_1 s_0 \theta_1 c_0 \theta_1 \right] \\
& \quad - \frac{d}{dt} \left[ J_{22}^* c^2 \theta_1 + J_{33}^* s^2 \theta_1 + 2 J_{23}^* s_0 \theta_1 c_0 \right] \dot{\theta}_2 - (J_{12}^* c_0 \theta_1 + J_{13}^* s_0 \theta_1) \dot{\theta}_1 \\
& \quad + (J_{23}^* c_0 + J_{33}^* s_0 \theta_1) \dot{\theta}_3 + K_2^* c_0 \theta_1 + K_3^* s_0 \theta_1 + \Omega \left[ -J_{12}^* c_0 \theta_1 s_0 \theta_1 \\
& \quad - J_{22}^* s_0 \theta_1 c_0 \theta_2 - J_{13}^* s_0 \theta_1 s_0 + J_{23}^* c_0 \theta_1 \left( c^2 \theta_1 - s^2 \theta_1 \right) c_0 \right. \\
& \left. \quad + J_{33}^* s_0 \theta_1 c_0 \theta_2 \right] = 0 \\
& \quad (108a)
\end{align*}
\]
Considering Eq. (36) in conjunction with Eqs. (69), (99), (100), and (102), and letting $i = 1$, we obtain the differential equation for $v_1$

$$\rho_1 \left( (h_{x_1} + x_1) \right) \left[ (\omega_1 c_1 + \omega_2 s_1)(\omega_1 s_1 + \omega_2 c_1) - 3\Omega^2 (\varepsilon_{a_1 c_1} + \varepsilon_{a_2 s_1})(\varepsilon_{a_1 s_1} - \varepsilon_{a_2 c_1}) \right] + (h_{x_1} + x_1) \left[ (\omega_1 c_1 + \omega_2 s_1)^2 + \omega_3^2 - 3\Omega^2 (\varepsilon_{a_1 c_1} + \varepsilon_{a_2 s_1})^2 \right] + \dot{\omega}_1 \left( ca_1 + sa_2 \right) + \frac{\partial}{\partial t} \rho_1 \left[ - (w_1 + h_{z_1})(\omega_1 c_1 + \omega_2 s_1) \right] + (h_{x_1} + x_1) \omega_3 + \dot{v}_1$$

$$- \frac{\partial}{\partial x_1} \left( - \Omega^2 (\varepsilon_{c_1 s_1} - \varepsilon_{c_2 c_1})^2 + \varepsilon_{c_3}^2 + 2 - 3[(\varepsilon_{a_1 s_1} - \varepsilon_{a_2 c_1})^2 \frac{\partial}{\partial t} \rho_1 \left[ - (w_1 + h_{z_1})(\omega_1 c_1 + \omega_2 s_1) \right] + (h_{x_1} + x_1) \omega_3 + \dot{v}_1$$

$$- \frac{\partial}{\partial x_1} \left( - \Omega^2 (\varepsilon_{c_1 s_1} - \varepsilon_{c_2 c_1})^2 + \varepsilon_{c_3}^2 + 2 - 3[(\varepsilon_{a_1 s_1} - \varepsilon_{a_2 c_1})^2 \right]$$

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which is subject to the boundary conditions

\[ v_1(0) = 0 \quad , \quad v_1(0) = 0 \]  

\[ m_1 \left\{ (h_{x1} + \varepsilon_1) \left[ (\omega_1 c_0 + \omega_2 s_0) (\omega_1 s_0 - \omega_2 c_0) - 3 \Omega^2 (\varepsilon_{a1 c_0} + \varepsilon_{a2 c_0}) \right] + \right. \\
\left. + \varepsilon_{a2 s_0} (\varepsilon_{a1 s_0} - \varepsilon_{a2 c_0}) \right\] + \left( h_{y1} + v_1 \right) \left[ (\omega_1 c_0 + \omega_2 s_0)^2 \right. \\
\left. + \omega_3^2 + 2 \Omega^2 - 3 \Omega^2 \left\langle (\varepsilon_{a1 c_0} + \varepsilon_{a2 s_0})^2 + \varepsilon_{a3}^2 \right\rangle \right] + \left( h_{z1} + \omega_1 \right) \left[ \omega_3 (\omega_1 s_0 - \omega_2 c_0) - 3 \Omega^2 \varepsilon_{a3} (\varepsilon_{a1 s_0} - \varepsilon_{a2 c_0}) \right] + \dot{\omega}_1 \left( c_0 \omega_1 + s_0 \omega_2 \right) \\
\left. \left. - \frac{3}{\varepsilon_0} \left[ -(w_1 + h_{z1}) (\omega_1 c_0 + \omega_2 s_0) + (h_{x1} + \varepsilon_1) \omega_3 + \dot{\omega}_1 \right] \right] \\
\left. - \Omega^2 \left\langle (\varepsilon_{c_1 s_0} - \varepsilon_{c_2 c_0})^2 + \varepsilon_{c3}^2 + 2 - 3[(\varepsilon_{a1 s_0} - \varepsilon_{a2 c_0})^2 + \varepsilon_{a3}^2] \right\rangle \right\} (h_{x1} + \varepsilon_1) v_1 + \frac{5}{2} E_1 v_1 v_1'' \\
\left. + \frac{a}{\varepsilon_{x1}} \left[ E_1 v_1'' (1 - \frac{5}{2} v_1'^2) \right] \left| x_1 = \varepsilon_1 = 0 \right. \right. \\
\left. E_1 v_1'' (1 - \frac{5}{2} v_1'^2) \left| x_1 = \varepsilon_1 = 0 \right. \right. \\
\left. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \right. \r
where \( w_1 \) must satisfy the boundary conditions

\[
\begin{align*}
  w_1(0) &= 0 , \quad w_1'(0) = 0 \\
  m_1 \left( (h_{x1} + \lambda_1) \right) \left[ -\omega_3(\omega_1 c_1 + \omega_2 s_1) + 3\Omega^2 \varepsilon_{a3} (\varepsilon_{a1} c_2 + \varepsilon_{a2} s_1) \right] \\
  &+ \left( h_{y1} + v_1 \right) \left[ \omega_3(\omega_1 s_1 - \omega_2 c_1) - 3\Omega^2 \varepsilon_{a3} (\varepsilon_{a1} s_2 - \varepsilon_{a2} c_1) \right] \\
  &+ \left( h_{z1} + w_1 \right) \left[ \omega_1^2 + \omega_2^2 + 2\Omega^2 - 3\Omega^2 \left( \varepsilon_{a1}^2 + \varepsilon_{a2}^2 \right) \right] \\
  &- \dot{v}_1 (\omega_1 c_1 + \omega_2 s_1) - \frac{\partial}{\partial t} \left[ (h_{x1} + \lambda_1) \left( \omega_1 s_1 - \omega_2 c_1 \right) + (h_{y1} + v_1) \left( \omega_1 c_1 + \omega_2 s_1 \right) + \omega_2 s_1 \right] + \dot{w}_1 \left\langle \left( \varepsilon_{c1} s_1 - \varepsilon_{c2} c_1 \right)^2 + \varepsilon_{c3}^2 + 2 - 3 \left[ \left( \varepsilon_{a1} s_1 - \varepsilon_{a2} c_1 \right)^2 + \varepsilon_{a3}^2 \right] \right\rangle (h_{x1} + \lambda_1) w_1' + \frac{5}{2} E I y_1 w_1' w_1'' \\
  &+ \frac{3}{\partial x_1} \left[ E I y_1 w_1'' \left( 1 - \frac{5}{2} \left( w_1' \right)^2 \right) \right] \bigg|_{x_1 = \lambda_1} = 0 \\
  E I y_1 w_1'' \left( 1 - \frac{5}{2} \left( w_1' \right)^2 \right) \bigg|_{x_1 = \lambda_1} = 0
\end{align*}
\]
The equations of motion and boundary conditions associated with the booms 2, 3, and 4 are obtained from Eqs. (109) and (110) by replacing \( \alpha \) by \( \pi - \alpha \), \( \pi + \alpha \), and \( 2\pi - \alpha \), respectively, and, of course, changing the subscripts of \( v_1 \) and \( w_1 \) accordingly.

Following the same procedure as that used to obtain Eqs. (109) and (110), the equation of motion and boundary conditions for \( v_5 \) are

\[
\rho_5 \left( (h_{x5} + x_5) [-\omega_1(\omega_2sb + \omega_3cb) + 3\Omega^2\epsilon_{a1}(\epsilon_{a2}sb + \epsilon_{a3}cb)] + (h_{y5} + v_5)[\omega_2^2 + \omega_3^2 + 2\Omega^2 - 3\Omega^2 (\epsilon_{a2}^2 + \epsilon_{a3}^2)] + (h_{z5} + w_5)[\omega_1(\omega_3sb
\]
- \omega_2 c_B - 3\Omega^2 \left( \ell_{a1} (\ell_{a1}s_B - \ell_{a2}s_B) \right) + (\omega_2 c_B + \omega_2 s_B)w_5 \right) \\
- \frac{3}{3t} \rho_5 \left[ (h_{x5} + x_5)(\omega_2 c_B - \omega_3 s_B) - (h_{z5} + w_5)(\omega_2 s_B + \omega_3 c_B) + \dot{v}_5 \right] \\
- \frac{3}{3x_5} \left( -\Omega^2 \left( \ell_{c1}^2 + (\ell_{c2} c_B - \ell_{c3} s_B)^2 \right) + 2 - 3[\ell_{a1} + (\ell_{a2} c_B - \ell_{a3} s_B)^2] \right) \\
\frac{1}{2} \rho_5 \left[ (h_{x5} + x_5)^2 - (h_{x5} + x_5)^2 \right] v_5^\prime + \frac{5}{2} EI_{z5} v_5^2 v_5^{''2} \\
\frac{3}{3x_5^2} \left[ -EI_{z5} v_5^\prime \left( 1 - \frac{5}{2} v_5^{'}2 \right) \right] + p_{y5} = 0 , \quad 0 < x_5 < z_5 \tag{112a}

v_5(0) = 0 , \quad v_5^{'}(0) = 0 \tag{112b}

\begin{align*}
\frac{5}{2} EI_{z5} v_5^{'} v_5^{''2} + \frac{3}{3x_5} \left[ EI_{z5} v_5^{''} \left( 1 - \frac{5}{2} v_5^{'}2 \right) \right] \bigg|_{x_5 = \ell_5} = 0
\end{align*} \tag{112c}

and those for w_5 are

\begin{align*}
\rho_5 \left[ (h_{x5} + x_5) \left[ (\omega_3 s_B - \omega_2 c_B)(\omega_2 s_B + \omega_3 c_B) - 3\Omega^2 (\ell_{a3}s_B - \ell_{a2}s_B)(\ell_{a2}s_B \\
+ \ell_{a3}s_B) \right] - (h_{y5} + v_5)[\omega_1(\omega_2 c_B - \omega_3 s_B) - 3\Omega^2 \ell_{a1}(\ell_{a2}c_B - \ell_{a3}s_B) \\
+ (h_{z5} + w_5) \left( \ell_{a2}s_B + \omega_3 c_B \right)^2 + 2\Omega^2 - 3\Omega^2 \left[ \ell_{a1} + (\ell_{a2}s_B \\
+ \ell_{a3}s_B)^2 \right] \right) \right] - \dot{v}_5 \left( \omega_2 s_B + \omega_3 c_B \right) \\
- \frac{3}{3t} \rho_5 \left[ -(h_{x5} + x_5)\omega_1 + (h_{y5} + v_5)(\omega_3 c_B + \omega_2 s_B) \right] \\
- \frac{3}{3x_5^2} \left( -\Omega^2 \left( \ell_{c1}^2 + (\ell_{c2} c_B - \ell_{c3} s_B)^2 \right) + 2 - 3[\ell_{a1} + (\ell_{a2} c_B - \ell_{a3} s_B)^2] \right) \\
\frac{1}{2} \rho_5 \left[ (h_{x5} + x_5)^2 - (h_{x5} + x_5)^2 \right] w_5^\prime + \frac{5}{2} EI_{y5} w_5^{'} w_5^{''2} \tag{112d}
\end{align*}
The equations for boom 6 are obtained from Eqs. (112) and (113) by an appropriate change in subscript, and by replacing \( \beta \) by \( \pi + \beta \).

b. Perturbation solution of the equilibrium problem.

The first problem in attempting a solution of the equations of motion, Eqs. (108), (109), (112), and (113), is to identify the equilibrium configurations. To this end, we must let all the velocities and accelerations equal to zero in these equations. This leaves us three transcendental equations for the rotations \( \theta_j \) \((j = 1, 2, 3)\) and twelve nonlinear differential equations for the elastic displacements \( v_i, w_i \) \((i = 1, 2, \ldots, 6)\).

We shall consider the solution of the nonlinear equilibrium problem in the form

\[
v_{i0}(x_i) = v_{i00}(x_i) + v_{i10}(x_i), \quad w_{i0}(x_i) = w_{i00}(x_i) + w_{i10}(x_i),
\]

\( i = 1, 2, \ldots, 6 \)

(114)

where the third subscripts on the right side of Eqs. (114) indicate the solution of the linearized problem if the subscript is zero and relatively
small perturbations if the subscript is one. It follows that the inertia matrix of the deformed body can be written as

$$[J(0)] = [J(0)]_0 + [J(0)]_1 = \sum_{i=0}^{6} [\varepsilon_i]^T [J_i]_0 [\varepsilon_i] + \sum_{i=1}^{6} [\varepsilon_i]^T [J_i]_1 [\varepsilon_i]$$  \hspace{1cm} (115)$$

where $[J(0)]_0$ is the inertia matrix as if the body was entirely rigid, in which

$$J_{0110} = A_0, \quad J_{0220} = B_0, \quad J_{0330} = C_0$$  \hspace{1cm} \text{(116)}$$

$$J_{0120} = J_{0210} = J_{0130} = J_{0310} = J_{0230} = J_{0320} = 0$$

are the moments of inertia of the rigid hub, and

$$J_{1110} = \int_0^{h_x} \rho_i (h_y^2 + h_z^2) dx_i + m_i (h_y^2 + h_z^2)$$

$$J_{1220} = \int_0^{h_x} \rho_i [(h_x + x_i)^2 + h_z^2] dx_i + m_i [(h_x + x_i)^2 + h_z^2]$$

$$J_{1330} = \int_0^{h_x} \rho_i [(h_x + x_i)^2 + h_y^2] dx_i + m_i [(h_x + x_i)^2 + h_y^2]$$

$$J_{1120} = J_{1210} = -\int_0^{h_x} \rho_i (h_x + x_i) h_y dx_i - m_i (h_x + x_i) h_y$$

$$J_{1130} = J_{1310} = -\int_0^{h_x} \rho_i (h_x + x_i) h_z dx_i - m_i (h_x + x_i) h_z$$

$$J_{1230} = J_{1320} = -\int_0^{h_x} \rho_i h_y h_z h_i dx_i - m_i h_y h_z$$

are the moments of inertia of the appendages when in undeformed state, expressed in terms of local coordinates. Moreover, $[J(0)]_1$ is the change
in the inertia matrix due to first order elastic displacements, which has
the elements

\[ J_{i 111} = \int_0^{x_i} \rho_i (2h_y i v_{100}^2 + v_{100}^2 + 2h_z i w_{i 100}^2 + w_{i 100}^2) dx_i \]

\[ + m_i (2h_y i v_{100}^2 + v_{100}^2 + 2h_z i w_{i 100}^2 + w_{i 100}^2) \big|_{x_i = \varepsilon_i} \]

\[ J_{i 221} = \int_0^{x_i} \rho_i (2h_z i w_{i 100}^2 + w_{i 100}^2) dx_i + m_i (2h_z i w_{i 100}^2 + w_{i 100}^2) \big|_{x_i = \varepsilon_i} \]

\[ J_{i 331} = \int_0^{x_i} \rho_i (2h_y i v_{100}^2 + v_{100}^2) dx_i + m_i (2h_y i v_{100}^2 + v_{100}^2) \big|_{x_i = \varepsilon_i} \]

\[ J_{i 121} = J_{i 211} = -\int_0^{x_i} \rho_i (h_{x_i} + x_i) v_{100} dx_i - m_i (h_{x_i} + x_i) v_{100} \big|_{x_i = \varepsilon_i} \]

\[ J_{i 131} = J_{i 311} = -\int_0^{x_i} \rho_i (h_{x_i} + x_i) w_{i 100} dx_i - m_i (h_{x_i} + x_i) w_{i 100} \big|_{x_i = \varepsilon_i} \]

\[ J_{i 231} = J_{i 321} = -\int_0^{x_i} \rho_i (h_y i w_{i 100} + h_z i v_{100} + v_{100} w_{i 100}) dx_i \]

\[ -m_i (h_y i w_{i 100} + h_z i v_{100} + v_{100} w_{i 100}) \big|_{x_i = \varepsilon_i} \]

\[ i = 1, 2, \ldots, 6 \] (118)

To linearize the algebraic equations for the angles \( \theta_j \) (\( j = 1, 2, 3 \))
we would have to assume that the angles are small. This, however, is not
always true for an arbitrary satellite, so that linearization cannot be
justified. Fortunately, it is not difficult to solve the nonlinear
algebraic equations for the angles \( \theta_j \) (\( j = 1, 2, 3 \)) by means of Newton-
Raphson method for the moments of inertia given. As a first iteration,
we insert the moments of inertia of the satellite regarded as rigid into
the three transcendental equations for $\theta_j$ ($j = 1, 2, 3$), and obtain some
preliminary values for these angles. Hence, letting all terms in Eqs.
(108) involving time derivatives equal to zero, we obtain

\[ 4J_{120}^* c_{2c_2}^2 (s_{2c_2} - 3s_{2c_2}) + 4J_{130}^* c_{1s_2}^2 s_{2c_2} - 4J_{230}^* (c_{2c_2}^2 - s_{2c_2}^2)(3s_{2c_2}^2 - c_{2c_2}^2) = 0 \]  

(119a)

\[ 4J_{110}^* s_{2c_2}^2 c_{2c_2} + 4J_{120}^* s_{1c_1}^2 (c_{2c_2}^2 - s_{2c_2}^2) - 4J_{220}^* s_{1s_2}^2 c_{2c_2} - 4J_{130}^* c_{1c_1}^2 (c_{2c_2}^2 - s_{2c_2}^2) \]

\[-s_{2c_2}^2) + 8J_{230}^* s_{1c_1} c_{1s_2} c_{2c_2} - 4J_{330}^* c_{2c_1} s_{1s_2} c_{2c_2} = 0 \]  

(119b)

\[ -J_{120}^* (s_{2c_2}^2 - 3c_{2c_2}^2) + 4(J_{110}^* - J_{220}^*) s_{1s_2} c_{2c_2} + J_{120}^* s_{2c_1} (c_{2c_2}^2 - 3s_{2c_2}^2) \]

\[ + 4J_{230}^* c_{1s_2} c_{2c_2} - J_{130}^* c_{1c_1} (c_{2c_2}^2 - 3s_{2c_2}^2) = 0 \]  

(119c)

where $[J^*]_0 = [\theta]^T [J(0)]_0 [\theta]$ in which $[\theta]$ is given by the first of
Eqs. (104). Regarding the angles $\theta_j$ ($j = 1, 2, 3$) as known constants, we
can linearize Eqs. (109), (110), (112), and (113) with respect to $v_i$, $w_i$
and their derivatives, and solve for the perturbed elastic displacements.
Hence, inserting Eqs. (114) into (109), we obtain the equations for $v_{100}$
in the form

\[ \rho_1 ((h_{x1} + x_1) [(\omega_{10} c_\alpha + \omega_{20} s_\alpha)(\omega_{10} s_\alpha - \omega_{20} c_\alpha) - 3\omega^2 (\varepsilon_{10} c_\alpha \varepsilon_{20} c_\alpha) \]

\[ + \varepsilon_{20} s_\alpha)(\varepsilon_{10} s_\alpha - \varepsilon_{20} c_\alpha)] + (h_{y1} + v_{100}) [(\omega_{10} c_\alpha + \omega_{20} s_\alpha)^2 + \omega^2_{30} \]

\[ + 2\omega^2 - 3\omega^2 (\varepsilon_{10} c_\alpha + \varepsilon_{20} s_\alpha)^2 + \varepsilon_{30}^2] + (h_{z1} + w_{100})[\omega_{30} (\omega_{10} s_\alpha \]

\[ + \varepsilon_{20} s_\alpha)] + (h_{w1} + w_{100}) [\omega_{30} (\omega_{10} s_\alpha \]

\[ + \varepsilon_{20} s_\alpha)] + (h_{w1} + w_{100}) [\omega_{30} (\omega_{10} s_\alpha \]

\[ + \varepsilon_{20} s_\alpha)] + (h_{w1} + w_{100}) [\omega_{30} (\omega_{10} s_\alpha \]
where \( v_{100} \) is subject to the boundary conditions

\[
v_{100}(0) = 0 \quad , \quad v_{100}'(0) = 0
\]  

\[
m_1 \left[ (h_{x1} + \xi_1) \left[ (\omega_{10c\alpha} + \omega_{20s\alpha})(\omega_{10s\alpha} - \omega_{20c\alpha}) - 3\Omega^2 (\lambda_{a10c\alpha} + \lambda_{a20s\alpha}) \right] + (h_{y1} + v_{100})[\omega_{10c\alpha} + \omega_{20s\alpha}]^2 \\ + \omega_{30}^2 + 2\Omega^2 - 3\Omega^2 \left( (\lambda_{a10c\alpha} + \lambda_{a20s\alpha})^2 + \lambda_{a30}^2 \right) + (h_{z1} + w_{100}) [\omega_{30}(\omega_{10s\alpha} - \omega_{20c\alpha}) - 3\Omega^2 \lambda_{a30}(\lambda_{a10s\alpha} - \lambda_{a20c\alpha})] \\ - v_{100}' \Omega^2 (h_{x1} + \xi_1) \left( (\lambda_{c10s\alpha} - \lambda_{c20c\alpha})^2 + \lambda_{c30}^2 + 2 - 3(\lambda_{a10s\alpha} - \lambda_{a20c\alpha})^2 + \lambda_{a30}^2 \right) \right] + EI v_{100}'' \bigg|_{x_1 = \xi_1} = 0 \right]
\]

\[
\text{EI } v_{100}'' \bigg|_{x_1 = \xi_1} = 0
\]

The quantities \( \omega_{j0} \), \( \lambda_{aj0} \) and \( \lambda_{cj0} \) (\( j = 1,2,3 \)) appearing in Eqs. (120) are to be calculated by using \( \theta_j \) (\( j = 1,2,3 \)) as given by Eqs. (119). Note that
now primes designate total derivatives with respect to $x_1$ because $v_{100}$ depends on $x$ alone and not on $t$. Similarly, for $w_{100}$ we have

$$
\rho_1 \left\{ (h_{x1} + x_1) \left[ -\omega_{30}(\omega_{10}c + \omega_{20}s) + 3\Omega^2\epsilon_{a30}(\epsilon_{a10}c + \epsilon_{a20}s) \right] \\
+ (h_{y1} + v_{100}) \left[ \omega_{30}(\omega_{10}c - \omega_{20}c) - 3\Omega^2\epsilon_{a30}(\epsilon_{a10}c - \epsilon_{a20}c) \right] \\
+ (h_{z1} + w_{100}) \left[ (\omega_1^2 + \omega_2^2) + 2\Omega^2 - 3\Omega^2(\epsilon_{a10}^2 + \epsilon_{a20}^2) \right] \right\} \\
- \rho_1 w_{100} \left( h_{x1} + x_1 \right) \Omega^2 \left( (\epsilon_{c10} + \epsilon_{c20}c) + \epsilon_{c30}^2 + 2 - 3[(\epsilon_{a10}^2 - \epsilon_{a20}c)^2 + \epsilon_{a30}^2] \right) + w_{100}'' \Omega^2 \left\{ \frac{1}{2} \rho_1 \left[ (h_{x1} + x_1)^2 - (h_{x1} + x_1) \right] \\
+ m_1 (h_{x1} + x_1) \left( (\epsilon_{c10} + \epsilon_{c20}c)^2 + \epsilon_{c30}^2 + 2 - 3[(\epsilon_{a10}^2 - \epsilon_{a20}c)^2 + \epsilon_{a30}^2] \right) - EI_{y1} w_{100}'' = 0 \right. \right. (121a)

where $w_{100}$ is subject to the boundary conditions

$$
\left\{ \begin{array}{l}
w_{100}(0) = 0 \\
, w_{100}'(0) = 0 \end{array} \right. (121b)

m_1 \left\{ (h_{x1} + x_1) \left[ -\omega_{30}(\omega_{10}c + \omega_{20}s) + 3\Omega^2\epsilon_{a30}(\epsilon_{a10}c + \epsilon_{a20}s) \right] \\
+ (h_{y1} + v_{100}) \left[ \omega_{30}(\omega_{10}c - \omega_{20}c) - 3\Omega^2\epsilon_{a30}(\epsilon_{a10}c - \epsilon_{a20}c) \right] \\
+ (h_{z1} + w_{100}) \left[ (\omega_1^2 + \omega_2^2) + 2\Omega^2 - 3\Omega^2(\epsilon_{a10}^2 + \epsilon_{a20}^2) \right] \right\} \\
- w_{100}' \Omega^2 \left( h_{x1} + x_1 \right) \left( (\epsilon_{c10} + \epsilon_{c20}c)^2 + \epsilon_{c30}^2 + 2 - 3[(\epsilon_{a10}^2 - \epsilon_{a20}c)^2 + \epsilon_{a30}^2] \right) \right\} + EI \left. w_{100}''' \right|_{x_1 = \epsilon_1} = 0 \right. \right. (121c)

$$
The differential equations and boundary conditions for \( v_{100} \) and \( w_{100} \) 
\( (i = 2, 3, 4) \) are obtained from Eqs. (120) and (121) by replacing the sub-
scripts of \( v_{100} \) and \( w_{100} \) by the appropriate ones and the angle \( \alpha \) by \( \pi - \alpha \),
\( \pi + \alpha \), and \( 2\pi - \alpha \), respectively. On the other hand, the differential
equations and boundary conditions for \( v_{500} \) and \( w_{500} \) are obtained from
Eqs. (112) and (113) in the form

\[
\rho_5 \left[ (h_x + x) \left[ -\omega_{10}(\omega_{20}s^2 + \omega_{30}c^2) + 3\Omega^2 \xi_{a10}(\xi_{a20}s^2 + \xi_{a30}c^2) \right] \right. \\
+ (h_y + v_{500}) \left[ \omega_{20}^2 + \omega_{30}^2 + 2\Omega^2 - 3\Omega^2 \xi_{a20} + \xi_{a30} \right] \\
+ (h_z + w_{500}) \left[ \omega_{10}(\omega_{30}s^2 - \omega_{20}c^2) - 3\Omega^2 \xi_{a10}(\xi_{a30}s^2 - \xi_{a20}c^2) \right] \\
\left. - \rho_5 \Omega^2 (h_x + x) \left\{ \xi_{c10}^2 + (\xi_{c20}c - \xi_{c30}s)^2 \right\} \right] \\
+ 2 - 3[\xi_{a10}^2 + (\xi_{a20}c - \xi_{a30}s)^2] + v''_{500} \Omega^2 \left\{ \frac{1}{2} \rho_5 \left[ (h_x + x) \right. \\
+ (h_y + y)^2 - (h_x + x)^2 \bigg\} \left\{ \xi_{c10}^2 + (\xi_{c20}c - \xi_{c30}s)^2 \right\} + 2 \\
- 3[\xi_{a10}^2 + (\xi_{a20}c - \xi_{a30}s)^2] \right\} - E_{Iz5} v''_{500} = 0
\]  

(122a)

where \( v_{500} \) satisfies the boundary conditions

\[
v_{500}(0) = 0 \quad , \quad v'_{500}(0) = 0
\]

(122b)

\[
E_{Iz5} v''_{500} \bigg|_{x_5 = z_5} = 0 \quad , \quad E_{Iz5} v''_{500} \bigg|_{x_5 = z_5} = 0
\]

(122c)

as well as

\[
\rho_5 \left[ (h_x + x) \left[ (\omega_{30}s^2 - \omega_{20}c^2)(\omega_{20}s^2 + \omega_{30}c^2) - 3\Omega^2 \xi_{a30}s^2
\right]
\]
The differential equations and boundary conditions for $v_{600}$ and $w_{600}$ are obtained by replacing in Eqs. (122) and (123) $v_{500}$ and $w_{500}$ by $v_{600}$ and $w_{600}$, respectively.

On the other hand, the boundary-value problem for the perturbation $v_{101}$ is defined by the differential equation

$$
\rho_1 v_{101} \{ (\omega_{10} c_0 + \omega_{20} c_0)^2 + \omega_{30} + 2\Omega^2 - 3\Omega^2 [ (\ell_{a10} c_0 + \ell_{a20} c_0)^2 + \ell_{a30}^2 ] \\
+ \rho_1 w_{101} [ \omega_{30} (\omega_{10} c_0 - \omega_{20} c_0) - 3\Omega^2 \ell_{a30} (\ell_{a10} c_0 - \ell_{a20} c_0) ] \\
+ v_{101}' \{ -\rho_1 \Omega^2 (h_1 + x_1) \left( \ell_{c10} c_0 - \ell_{c20} c_0 \right) + \ell_{c30}^2 + 2 
$$
- 3[(e_{a10s} - e_{a20c})^2 + \frac{1}{2} \lambda_{a30}] + EI_{Z1}[10 v_{100}'' v_{100} + 5 v_{100}'' v_{100}]

+ v_{101}'' \ (\frac{1}{2} \rho_1 [h_{x1} + \lambda_1] - (h_{x1} + x_1)^2] + m_1 (h_{x1}

+ \lambda_1) \left( (e_{c10s} - e_{c20c}) + \frac{1}{2} \lambda_{c30} + 2 - 3[(e_{a10s} - e_{a20c})^2

+ \frac{1}{2} \lambda_{a30}] + EI_{Z1} \left[ \frac{15}{2} (v_{100}''')^2 + 10 \ v_{100}'' v_{100}' \right] \right)

+ EI_{Z1} \ v_{101}'' (10 v_{100}'' v_{100}'') - EI_{Z1} \ v_{101}'' [1 - \frac{5}{2} (v_{100}')^2] = -EI_{Z1} \left[ \frac{5}{2} (v_{100}')^3

+ 10 v_{100}' v_{100}'' v_{100}'' + \frac{5}{2} v_{100}'' (v_{100}')^2 \right]

(124a)

and the boundary conditions

\[ v_{101}(0) = 0 \ , \ v_{101}'(0) = 0 \] (124b)

\[ m_1 v_{101} \ (\omega_{10c} + \omega_{20s})^2 + \omega_{30}^2 + 2 \Omega^2 - 3\Omega^2 \left( (e_{a10s} - e_{a20c})^2 + \frac{1}{2} \lambda_{a30} \right) + \]

\[ m_1 w_{101} \ [\omega_{30} (\omega_{10s} - \omega_{20c}) - 3\Omega^2 e_{a30} (e_{a10s} - e_{a20c})] \]

\[ + v_{101}'' \ (m_1 \Omega^2 (h_{x1} + \lambda_1) - \frac{1}{2} \lambda_{a30} + 2 \]

\[ - 3[(e_{a10s} - e_{a20c})^2 + \frac{1}{2} \lambda_{a30}] \right) - \frac{5}{2} EI_{Z1} \left[ 2v_{100}' v_{100}'' + (v_{100}')^2 \right] \]

\[ - EI_{Z1} \ v_{101}'' (5v_{100} v_{100}') \]

\[ + EI_{Z1} \ v_{101}'' [1 - \frac{5}{2} (v_{100}')^2] = \frac{5}{2} EI_{Z1} \ [(v_{100}')^2 v_{100}'' \]

\[- 2v_{100}' (v_{100}')^2] \]

\[- [EI_{Z1} \ [(1 - \frac{5}{2} (v_{100}')^2) v_{101}'' \]

\[- 5v_{100}' v_{100}'' v_{101}'] = -\frac{5}{2} EI_{Z1} \ (v_{100}')^2 v_{100}'' \]

at \( x_1 = \lambda_1 \) (124c)
and that for the perturbation \( w_{101} \) by

\[
\rho_1 y_{101} \left[ \omega_{30}(\omega_{10}^{s\alpha} - \omega_{20}^{c\alpha}) - 3\Omega^2 \epsilon_{a30}(\epsilon_{a10}^{s\alpha} - \epsilon_{a20}^{c\alpha}) \right] \\
+ \rho_1 w_{101} \left[ \omega_{10}^2 + \omega_{20}^2 + 2\omega^2 - 3\Omega^2(\epsilon_{a10}^2 + \epsilon_{a20}^2) \right] \\
+ w_{101} \left\{ -\rho_1 \Omega^2 (h_{x1} + x_1) \left\langle (\epsilon_{c10}^{s\alpha} - \epsilon_{c20}^{c\alpha}) + \epsilon_{c30}^2 + 2 \\
- 3[(\epsilon_{a10}^{s\alpha} - \epsilon_{a20}^{c\alpha})^2 + \epsilon_{a30}^2] \right\rangle \right.
+ \left[ 10 w_{100}^2 w_{100}^* \\
+ 5 w_{100}^* w_{100} \right] \left( w_{100}^* \right)^2 \right\} + E I y_1 \left[ \frac{15}{2} (w_{100}^* w_{100})^2 + 10 w_{100}^* w_{100} w_{100}^* \right] \\
+ E I y_1 w_{101}^* (10 w_{100}^* w_{100}^*) \\
- E I y_1 \left[ 1 - \frac{5}{2} (w_{100}^* w_{100})^2 \right] = - E I y_1 \left[ \frac{5}{2} (w_{100}^* w_{100})^3 + 10 w_{100}^* w_{100} w_{100}^* w_{100}^* \right] \\
+ \frac{5}{2} w_{100}^* \left( w_{100}^* \right)^2 \\
\tag{125a}
\]

\[
w_{101}(0) = 0 \quad w_{101}'(0) = 0 \quad \tag{125b}
\]

\[
m_1 y_{101} \left[ \omega_{30}(\omega_{10}^{s\alpha} - \omega_{20}^{c\alpha}) - 3\Omega^2 \epsilon_{a30}(\epsilon_{a10}^{s\alpha} - \epsilon_{a20}^{c\alpha}) \right] \\
+ m_1 w_{101} \left[ \omega_{10}^2 + \omega_{20}^2 + 2\omega^2 - 3\Omega^2(\epsilon_{a10}^2 + \epsilon_{a20}^2) \right] \\
+ w_{101} \left\{ m_1 \Omega^2 (h_{x1} + x_1) \left\langle (\epsilon_{c10}^{s\alpha} - \epsilon_{c20}^{c\alpha}) + \epsilon_{c30}^2 + 2 - 3[(\epsilon_{a10}^{s\alpha} \\
- \epsilon_{a20}^{c\alpha})^2 + \epsilon_{a30}^2] \right\rangle - \frac{5}{2} E I y_1 \left[ 2w_{100}^* w_{100} + (w_{100}^*)^2 \right] \\
\right.
\tag{125c}
\]

66
\(- EIy_1 \frac{w''_{\text{101}} (5w'_{\text{100}} w''_{\text{100}})}{2w'_{\text{100}} (w''_{\text{100}})^2} \bigg|_{x_5 = \lambda_5} \)

\(- EIy_1 \left[ (1 - \frac{5}{2} (w'_{\text{10}})^2 \right] w''_{\text{101}} \right) \right)

\(- 5w'_{\text{100}} w''_{\text{100}} w'_{\text{101}} = - \frac{5}{2} EIy_1 (w'_{\text{100}})^2 w''_{\text{100}} \bigg|_{x_5 = \lambda_5} \)

\text{(125c)}

with companion equations for \(v_{\text{101}}\) and \(w_{\text{101}}\) \((i = 2, 3, 4)\). In a like manner

\[ a_5 v_{501} \left[ \omega_{20}^2 + \omega_{30}^2 + 2\Omega^2 - 3\Omega^2 (\lambda_{a20}^2 + \lambda_{a30}^2) \right] \]

\[ + a_5 w_{501} \left[ \omega_{10} (\omega_{30}\beta - \omega_{20}\beta) - 2\Omega^2 (\lambda_{a10}^2 - \lambda_{a20}\beta) \right] \]

\[ + v_{501} \left[ -a_5 \Omega^2 (h_{x5} + x_5)^2 + (\lambda_{c10}^2 + (\lambda_{c20} + \lambda_{c30}\beta)^2) \right] + EIz_5 \left[ 10 v''_{500} v''_{500} + 5 v''_{500} v''_{500} \right] \]

\[ + v''_{501} \left[ a_5 \Omega^2 \left[ (h_{x5} + x_5)^2 - (h_{x5} + x_5)^2 \right] \right] \left[ \lambda_{c10}^2 + (\lambda_{c20} + \lambda_{c30}\beta)^2 \right] \right] + EIz_5 \left[ \frac{15}{2} (v''_{500})^2 \right.

\[ + 10 v_{500} v''_{500} \right] + EIz_5 v''_{501} \left( 10 v_{500} v''_{500} \right) \]

\[ - EIz_5 v''_{501} \left[ 1 - \frac{5}{2} (v_{500})^2 \right] = - \frac{5}{2} \left( v_{500} \right)^3 \]

\[ + 10 v_{500} v''_{500} v''_{500} + \frac{5}{2} v''_{500} (v''_{500})^2 \right] \]

\text{(126a)}

\[ v_{501}(0) = 0 \quad v'_{501}(0) = 0 \]

\text{(126b)}
\[ EI_{z5} \{- \frac{5}{2} \left[ 2 v_{500}v_{500}'' + (v_{500}'')^2 \right] v_{501} - 5 v_{500}v_{500}''v_{501} \\
+ \left[ 1 - \frac{5}{2} (v_{500}')^2 \right] v_{501}' \} \\
= \frac{5}{2} EI_{z5} \left[ (v_{500}')^2 v_{500}'' + 2 v_{500}' (v_{500}'')^2 \right] \\
- EI_{z5} \left[ \left[ (1 - \frac{5}{2} (v_{500}')^2 \right] v_{501}'' \right] \right] \text{ at } x_5 = \xi_5 \quad (126c) \]

and

\[ \alpha_5 v_{501}' [\alpha_{10} (\omega_2 \omega_3 - \omega_3 \omega_2) - 3 \Omega^2 \xi_{a10} (\omega_2 \omega_3 - \omega_3 \omega_2)'] \\
+ \rho_5 w_{501} \left\{ \frac{1}{2} \Omega^2 \left[ (h_{x5} + \xi_5)^2 - (h_{x5} - \xi_5)^2 \right] \right\} \left\{ \left[ \alpha_{10} + (\xi_{a20} - \xi_{a30}) \right] \right\} \\
+ EI_{y5} [10 w_{500}''w_{500}'' + 5 w_{500}'w_{500}''] \\
+ w_{501}'' \left\{ \frac{1}{2} \Omega^2 \left[ (h_{x5} + \xi_5)^2 - (h_{x5} - \xi_5)^2 \right] \right\} \left\{ \left[ \alpha_{10} + (\xi_{c20} - \xi_{c30}) \right] \right\} \\
+ 10 w_{500}'w_{500}'' \right\} + EI_{y5} w_{501}'' \left( 10 w_{500}''w_{500}' \right) \\
- EI_{y5} w_{501}'' \left[ \frac{5}{2} (w_{500}')^2 \right] = - EI_{y5} \left[ \frac{5}{2} (w_{500}'')^3 \right] \\
+ 10 w_{500}'w_{500}''w_{500}' + \frac{5}{2} w_{500}'' (w_{500}')(2) \right\} \quad (127a) \]

\[ w_{501}(0) = 0 \quad , \quad w_{501}'(0) = 0 \quad (127b) \]
It should be pointed out that this particular perturbation scheme enables us to solve first Eqs. (119) for the first approximation rotations $\theta_{jo0}$ ($j = 1,2,3$) independently of the elastic displacements. The rotations are then introduced into Eqs. (120) - (123), yielding the first approximation for the elastic displacements $v_{io0}$ and $w_{io0}$ ($i = 1,2,...,6$) independently of one another. Inserting the first approximation $v_{io0}$ and $w_{io0}$ into Eqs. (124) - (127), we can obtain the corrections $v_{i10}$ and $w_{i10}$ to the elastic displacements. The sums of these solutions yield $v_{i0}$ and $w_{i0}$ ($i = 1,2,...,6$) according to Eqs. (114). Then, inserting $v_{i0}$ and $w_{i0}$ ($i = 1,2,...,6$) back into Eqs. (119), we obtain the angles $\theta_{jo}$ ($j = 1,2,3$). In the vast majority of cases, this approximation is sufficient. If not, having the new angles, we can iterate once more to improve the elastic displacements $v_{i0}$ and $w_{i0}$, as well as the angles $\theta_{jo}$.

**c. Liapunov stability analysis and the eigenvalue problem.**

The values $\theta_{jo}$, $v_{i0}$, and $w_{i0}$ obtained above, together with sets of admissible functions $\phi_j(x_i)$ and $\psi_j(x_i)$, are subsequently introduced into Eqs. (49), (51), and (53), to obtain the coefficients $m_{jk}$, $f_{jk}$, and...
The coefficients $m_{jk}$ and $k_{jk}$ yield the symmetric matrices $[m]$ and $[k]$, whereas using Eq. (57) the coefficients $f_{jk}$ yield the skew symmetric matrix $[g]$.

From Sec. 6, if $[k]$ represents a positive definite matrix, then the nontrivial equilibrium is asymptotically stable. On the other hand, to obtain the natural frequencies, we must solve the eigenvalue problem in the form (66). However, before the nontrivial equilibrium can be determined, the system stability tested, and the natural frequencies calculated, it is desirable to use specific values for the system parameters. This is done in the next section.

d. The shortening of the projections effect

As indicated in Sec. 2, the booms are assumed to be inextensional, so that there is no longitudinal vibration. However, because of the transverse displacements, there is a shortening of the projection on the nominal axis of any element of length of the boom. In fact, from Eq. (14), the change in length of projection of any element of length $dx_i$ is

$$du_i = -\frac{1}{2} \left[ \left( \frac{\partial v_i}{\partial x_i} \right)^2 + \left( \frac{\partial w_i}{\partial x_i} \right)^2 \right] dx_i , \quad i = 1, 2, \ldots, 6 \quad (128)$$

We shall treat this shortening as a perturbation of the spatial coordinate $x_i$, so that we can write

$$x_i = x_{i0} + x_{i1} , \quad 0 \leq x_{i0} \leq l_i , \quad i = 1, 2, \ldots, 6 \quad (129)$$

where $x_{i0}$ are the original spatial coordinates and $x_{i1}$ are the
perturbations. From Eqs. (128), however, we conclude that the shortening is a second-order effect. Hence, it will not affect Eqs. (120) - (123) except that \( x_i \) are to be regarded in these equations as \( x_{i0} \) \((i = 1, 2, \ldots, 6)\). This enables us to solve for \( v_{100} \) and \( w_{100} \) and write the shortening of the projections in the form

\[
x_{i1} = \int_0^{x_{i0}} du_i = - \frac{1}{2} \int_0^{x_{i0}} \left[ \left( \frac{\partial v_{100}}{\partial \xi_i} \right)^2 + \left( \frac{\partial w_{100}}{\partial \xi_i} \right)^2 \right] d\xi_i,
\]

\[
i = 1, 2, \ldots, 6
\]

where \( \xi_i \) is a dummy variable. On the other hand, the perturbation equations, Eqs. (124) - (127), must be modified to account for the shortening effect. For example, the boundary-value problem for \( v_{101} \) becomes

\[
\rho_1 v_{101} \{(\omega_{10}c_a + \omega_{20}s_a)^2 + \omega_{30}^2 + 2\eta^2 - 3\eta^2[(\ell_{a10}c_a + \ell_{a20}s_a)^2 + \ell_{a30}^2]\}
\]

\[
+ \rho_1 w_{101} [\omega_{30}(\omega_{10}c_a - \omega_{20}s_a) - 3\eta^2\ell_{a30} (\ell_{a10}c_a - \ell_{a20}s_a)]
\]

\[
+ v_{101} \left[ - \rho_1 \Omega^2 (h_{x1} + x_{10}) \left( (\ell_{c10}s_a - \ell_{c20}c_a) + \ell_{c30}^2 + 2 - 3[\ell_{a10}s_a - \ell_{a20}c_a]^2 + \ell_{a30}^2] \right) + E_{Iz1} (10 v_{100}'' v_{100} + 5 v_{100} v_{100}') \right]
\]

\[
+ v_{101}'' \left\{ \Omega^2 \left( \frac{1}{2} \rho_1 [(h_{x1} + x_{10})^2 - (h_{x1} + x_{10})^2] + m_1 (h_{x1} + x_{10}) \right) \right. \\
\left. \left. + \ell_{10} \right\} \left( (\ell_{c10}s_a - \ell_{c20}c_a)^2 + \ell_{c30}^2 + 2 - 3[(\ell_{a10}s_a - \ell_{a20}c_a)^2 + \ell_{a30}^2] \right) \right. \\
\left. + E_{Iz1} \left[ \frac{15}{2} (v_{100}'')^2 + 10 v_{100} v_{100}''' \right] + E_{Iz1} (v_{101}''' (10 v_{100} v_{100}') \right.
\]

\[
- v_{101}' \left[ 1 - \frac{5}{2} (v_{100}')^2 \right] - E_{Iz1} \left[ \frac{5}{2} (v_{100}'')^2 + 10 v_{100} v_{100}' v_{100}'' \right.
\]

\[
+ \frac{5}{2} v_{100}'' (v_{100}')^2 + \frac{1}{2} \rho_1 [(\omega_{10}c_a + \omega_{20}s_a)(\omega_{10}s_a - \omega_{20}c_a)]
\]

\[
= 0.
\]
subject to boundary conditions (125b) and (125c), where in the latter \( \xi_1 \) must be replaced by the shortened length \( \xi_{10} \).


The general formulation of Sec. 11 has been used to obtain the nontrivial equilibrium configuration of the RAE/B satellite, to test the stability of equilibrium, and to calculate the natural frequencies of oscillation about the nontrivial equilibrium. The system parameters are as follows:

\[
\begin{align*}
A_0 &= 87.74 \text{ slug ft}^2, \quad B_0 = 83.74 \text{ slug ft}^2, \quad C_0 = 18 \text{ slug ft}^2 \\
\rho_1 &= \rho_2 = \rho_3 = \rho_4 = 4.348 \times 10^{-4} \text{ slug ft}^{-1}, \quad \rho_5 = \rho_6 = 4.596 \\
&\quad \times 10^{-4} \text{ slug ft}^{-1} \\
m_1 &= m_2 = m_3 = m_4 = 2.40 \times 10^{-3} \text{ slug}, \quad m_5 = m_6 = 0 \\
\xi_1 &= \xi_2 = \xi_3 = \xi_4 = 600 \text{ ft}, \quad \xi_5 = \xi_6 = 315 \text{ ft} \\
EI_{y1} &= EI_{z1} = EI_{y2} = \ldots = EI_{z4} = 15.278 \text{ lb ft}^2, \quad \alpha = 30^\circ \\
EI_{y5} &= EI_{z5} = EI_{y6} = EI_{z6} = 13.889 \text{ lb ft}^2, \quad \beta = 25^\circ \\
h_{x1} &= h_{x4} = 0.973 \text{ ft}, \quad h_{x2} = h_{x3} = 0.878 \text{ ft}, \quad h_{x5} = h_{x6} = 0 \\
h_{y1} &= -h_{y4} = 0.705 \text{ ft}, \quad h_{y2} = -h_{y3} = -0.760 \text{ ft}, \quad h_{y5} = h_{y6} = -1.800 \text{ ft} \\
h_{z1} &= h_{z2} = h_{z3} = h_{z4} = h_{z5} = h_{z6} = 0 \\
\gamma &= 4.653 \times 10^{-4} \text{ rad sec}^{-1}
\end{align*}
\]

We shall present the results of the analyses in the order listed above.
a. Nontrivial equilibrium

Inserting the above data into Eqs. (116) and (117), as indicated in Eq. (115), and solving Eqs. (119), we obtain

\[ \theta_{100} = 0.13537 \text{ rad} = 7.756 \text{ deg.} \]
\[ \theta_{200} = -5.63789 \times 10^{-8} \text{ rad} = -3.2302 \times 10^{-6} \text{ deg.} \]
\[ \theta_{300} = 1.37374 \times 10^{-6} \text{ rad} = 7.87096 \times 10^{-5} \text{ deg.} \]

We note that \( \theta_{100} \) is caused largely by the damper booms. The fact that the rods are not attached at the satellite mass center turns out to have an insignificant effect on \( \theta_{j00} \) (j = 1, 2, 3).

To evaluate the elastic displacements \( v_{100}(x_i) \) and \( w_{100}(x_i) \) (i = 1, 2, ..., 6). We assume the solution of Eqs. (120) - (123) in the form

\[
v_{100}(x_i) = \sum_{r=1}^{p} a_{r10} \phi_r(x_i) \quad i = 1, 2, ..., 6 
\]

\[
w_{100}(x_i) = \sum_{r=1}^{p} b_{r10} \phi_r(x_i) 
\]

where

\[
\phi_r(x_i) = A_r [(\cos \beta_r \lambda_i + \cosh \beta_r \lambda_i)(\sin \beta_r x_i - \sinh \beta_r x_i)
- (\sin \beta_r \lambda_i + \sinh \beta_r \lambda_i)(\cos \beta_r x_i - \cosh \beta_r x_i)]
\]

are eigenfunctions corresponding to a bar in bending with the end \( x_i = 0 \) fixed and having a mass \( m_i \) attached at the end \( x_i = \lambda_i \). The eigenvalues \( \beta_r \lambda_i \) are solutions of the characteristic equation
Moreover, the amplitudes $A_i$ are such that the eigenfunctions $\phi_r(x_i)$ are orthonormal, i.e., they satisfy the relation

$$\int_0^{2\pi} \rho_i \phi_r(x_i) \phi_s(x_i) dx_i + m_i \phi_r(x_i) \phi_s(x_i) = \delta_{rs}$$

where $\delta_{rs}$ is the Kronecker delta. Limiting the series in (132) to two terms, $p = 2$, the first two roots of Eq. (134) and the amplitudes $A_i$ corresponding to $i = 1,2,3,4$ are

$$\begin{align*}
\beta_1 \ell_i &= 1.85813 & A_1 &= 0.47696 \text{ slug}^{-1/2} \\
\beta_2 \ell_i &= 4.65310 & A_2 &= 0.03789 \text{ slug}^{-1/2}
\end{align*}$$

In addition, the coefficients $a_{ri0}$, $b_{ri0}$ ($i = 1,2,3,4$) are

<table>
<thead>
<tr>
<th></th>
<th>$a_{1i0}$</th>
<th>$a_{2i0}$</th>
<th>$b_{1i0}$</th>
<th>$b_{2i0}$</th>
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</thead>
<tbody>
<tr>
<td>1</td>
<td>$-0.13656 \times 10^2$</td>
<td>$-0.98055 \times 10^{-1}$</td>
<td>0.55184</td>
<td>0.46385 $\times 10^{-2}$</td>
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<tr>
<td>2</td>
<td>$0.13652 \times 10^2$</td>
<td>$0.97981 \times 10^{-1}$</td>
<td>0.55188</td>
<td>0.46385 $\times 10^{-2}$</td>
</tr>
<tr>
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<td>$-0.13652 \times 10^2$</td>
<td>$-0.97980 \times 10^{-1}$</td>
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<td>-0.46384 $\times 10^{-2}$</td>
</tr>
<tr>
<td>4</td>
<td>$0.13656 \times 10^2$</td>
<td>$0.98054 \times 10^{-1}$</td>
<td>-0.55184</td>
<td>-0.46385 $\times 10^{-2}$</td>
</tr>
</tbody>
</table>

The first two roots of Eq. (134) with $m_i = 0$, and the amplitudes $A_1$ and $A_2$ corresponding to $i = 5,6$ are
\[ B_{1\ell_1} = 1.87511 \quad A_1 = 0.63510 \quad \text{slug}^{-1/2} \]

\[ B_{2\ell_1} = 4.69414 \quad A_2 = 0.04899 \quad \text{slug}^{-1/2} \]

whereas the coefficients \( a_{ri0} \) and \( b_{ri0} \) are

Table II

<table>
<thead>
<tr>
<th>( i )</th>
<th>( a_{1i0} )</th>
<th>( a_{2i0} )</th>
<th>( b_{1i0} )</th>
<th>( b_{2i0} )</th>
</tr>
</thead>
<tbody>
<tr>
<td>5</td>
<td>-0.93855 \times 10^{-2}</td>
<td>-0.12900 \times 10^{-3}</td>
<td>0.17756</td>
<td>0.70688 \times 10^{-3}</td>
</tr>
<tr>
<td>6</td>
<td>-0.93840 \times 10^{-2}</td>
<td>-0.12900 \times 10^{-3}</td>
<td>0.17756</td>
<td>0.70688 \times 10^{-3}</td>
</tr>
</tbody>
</table>

It will prove of interest to list the elastic displacements of the end points, as calculated by means of the linearized equations. These displacements are

\[ v_{100}(x_1) = -52.205 \text{ ft}, \quad w_{100}(\ell_1) = 2.1071 \text{ ft}, \]
\[ v_{200}(\ell_2) = 52.192 \text{ ft}, \quad w_{200}(\ell_2) = 2.1072 \text{ ft}, \]
\[ v_{300}(\ell_3) = -52.191 \text{ ft}, \quad w_{300}(\ell_3) = -2.1072 \text{ ft}, \]
\[ v_{400}(\ell_4) = 52.204 \text{ ft}, \quad w_{400}(\ell_4) = -2.1071 \text{ ft}, \]
\[ v_{500}(\ell_5) = -4.8655 \times 10^{-2} \text{ ft}, \quad w_{500}(\ell_5) = 0.92961 \text{ ft}, \]
\[ v_{600}(\ell_6) = -4.8648 \times 10^{-2} \text{ ft}, \quad w_{600}(\ell_6) = 0.92960 \text{ ft} \]

The above values of \( v_{100}(x_i) \) and \( w_{100}(x_i) \) \((i = 1, 2, \ldots, 6)\) enable us to solve Eqs. (119) for the angles \( \varrho_{j0} \) \((j = 1, 2, 3)\) and Eq. (131) and the companion ones for the perturbations \( v_{101}(x_i) \), \( w_{101}(x_i) \) \((i = 1, 2, \ldots, 6)\). The resulting angles are
\[ \theta_{10} = 0.19695 \text{ rad} = 11.2846 \text{ deg.} \]
\[ \theta_{20} = -6.54250 \times 10^{-8} \text{ rad} = -3.74858 \times 10^{-6} \text{ deg.} \]
\[ \theta_{30} = 1.37608 \times 10^{-9} \text{ rad} = 7.88438 \times 10^{-8} \text{ deg.} \]

Instead of listing the perturbations \( v_{101} \) and \( w_{101} \), we shall list the complete solutions \( v_{i0} \) and \( w_{i0} \) in the form of the series

\[
v_{i0}(x_i) = \sum_{r=1}^{2} a_{ri} \phi_r(x_i)
\]

\[ i = 1, 2, \ldots, 6 \quad (138) \]

\[
w_{i0}(x_i) = \sum_{r=1}^{2} b_{ri} \phi_r(x_i)
\]

where \( \phi_r(x_i) \) are still given by Eqs. (133), in which the eigenvalues \( \beta_{ri} \) and amplitudes \( A_r \) (\( r = 1, 2 \)) are given by (136) and (137). The final results are tabulated as follows

Table III

<table>
<thead>
<tr>
<th>( i )</th>
<th>( a_{1i} )</th>
<th>( a_{2i} )</th>
<th>( b_{1i} )</th>
<th>( b_{2i} )</th>
</tr>
</thead>
<tbody>
<tr>
<td>1</td>
<td>(-0.13803 \times 10^2)</td>
<td>(-0.86379 \times 10^{-1})</td>
<td>0.54865</td>
<td>0.46378 \times 10^{-2}</td>
</tr>
<tr>
<td>2</td>
<td>0.13800 \times 10^2</td>
<td>0.86313 \times 10^{-1}</td>
<td>0.54869</td>
<td>0.46378 \times 10^{-2}</td>
</tr>
<tr>
<td>3</td>
<td>(-0.13800 \times 10^2)</td>
<td>(-0.86313 \times 10^{-1})</td>
<td>-0.54869</td>
<td>-0.46378 \times 10^{-2}</td>
</tr>
<tr>
<td>4</td>
<td>0.13803 \times 10^2</td>
<td>0.86379 \times 10^{-1}</td>
<td>-0.54865</td>
<td>-0.46378 \times 10^{-2}</td>
</tr>
<tr>
<td>5</td>
<td>(-0.93855 \times 10^{-2})</td>
<td>(-0.12900 \times 10^{-3})</td>
<td>0.17756</td>
<td>0.70670 \times 10^{-3}</td>
</tr>
<tr>
<td>6</td>
<td>(-0.93840 \times 10^{-2})</td>
<td>(-0.12900 \times 10^{-3})</td>
<td>0.17756</td>
<td>0.70670 \times 10^{-3}</td>
</tr>
</tbody>
</table>

76
Moreover, the final end displacements are

\[ v_{10}(\theta_1) = -52.816 \text{ ft,} \]
\[ w_{10}(\theta_1) = 2.0948 \text{ ft,} \]
\[ v_{20}(\theta_2) = 52.803 \text{ ft,} \]
\[ w_{20}(\theta_2) = 2.0950 \text{ ft,} \]
\[ v_{30}(\theta_3) = -52.803 \text{ ft,} \]
\[ w_{30}(\theta_3) = -2.0950 \text{ ft,} \]
\[ v_{40}(\theta_4) = 52.816 \text{ ft,} \]
\[ w_{40}(\theta_4) = -2.0948 \text{ ft,} \]
\[ v_{50}(\theta_5) = -4.8655 \times 10^{-2} \text{ ft,} \]
\[ w_{50}(\theta_5) = 0.92962 \text{ ft,} \]
\[ v_{60}(\theta_6) = -4.8648 \times 10^{-2} \text{ ft,} \]
\[ w_{60}(\theta_6) = 0.92962 \text{ ft,} \]

and we note that the nonlinear effect is virtually zero for booms 5 and 6. The nontrivial equilibrium is depicted in Fig. 7, where only the radial booms are shown because the displacements of the damper booms are insignificant.

b. Liapunov stability analysis

A stability analysis using \( \kappa \), as given by Eq. (60), as a testing function has been carried out. Essentially, the analysis reduced to testing the matrix \([k]\) for positive definiteness, where the elements of \([k]\) are given by Eqs. (53). The numerical values of the elements for the particular configuration at hand are listed in the next subsection. The matrix was found to be positive definite, so that the equilibrium is asymptotically stable.

c. Eigenvalue problem

Using Eqs. (49), (51), (53), and (57), in conjunction with the above data, we obtain the elements
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<tr>
<th>$m_1,j$</th>
<th>$3.42090 \times 10^4$</th>
<th>$0$</th>
<th>$-6.31727 \times 10^{-3}$</th>
<th>$4.78807 \times 10^{-1}$</th>
<th>$-7.67484 \times 10$</th>
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<tr>
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<td>$-4.78845 \times 10^{-1}$</td>
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| \( 0 \) | 0 | 0 | 0 | 0 | 0 |
| \( 0 \) | 3.71129 \times 10^{-5} | 8.10039 \times 10^{-13} | 0 | 0 | 0 |

| \( k_{13,j} \) | \(-4.36481 \times 10^{-6} \) | \(-2.79445 \times 10^{-7} \) | \(-3.34538 \times 10^{-7} \) | 0 | 0 | 0 |
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| \( k_{15,j} \) | \(-4.36488 \times 10^{-6} \) | \(2.79513 \times 10^{-7} \) | \(3.34444 \times 10^{-7} \) | 0 | 0 | 0 |
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Note that the elastic displacements were represented by one mode each.

Using the formulation of Section 7, we obtain the following natural frequencies

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The natural modes have also been obtained and will be discussed in a future paper.

d. Parametric study

The stability analysis was carried one step farther by varying the angle $\alpha$. It was found that the system was asymptotically stable for $\alpha = 50^\circ$, but became unstable for $\alpha = 51^\circ$. The results can be easily explained by the fact that in the absence of damper booms and for completely rigid radial booms the system becomes unstable around $\alpha = 45^\circ$. The gravitational and centrifugal effects tend to deform the flexible booms in a manner that the moments of inertia about the local vertical and
about an axis tangent to the orbit are the same for an angle \( \alpha \) such that \( 50^\circ < \alpha < 51^\circ \). It should be mentioned that instability in both cases can be traced to angle \( \theta_3 \), which tends to become large when the moment of inertia about the local vertical becomes larger than that about the axis tangent to the orbit, as at this point the "least moment of inertia" criterion is violated.

The same parametric study was undertaken with respect to the natural frequencies. In terms of natural frequencies, instability occurs when at least one natural frequency (we recall that in our case the natural frequencies occur in pairs) reduces to zero. Here again the system becomes unstable for \( 50^\circ < \alpha < 51^\circ \), thus corroborating the results obtained by the Liapunov stability analysis.

13. Summary and Conclusions

Two new theories for studying the motion characteristics of a rotating system with flexible parts about undeformed equilibrium have been developed. The first is qualitative and the second quantitative. Specifically, the first represents a stability theory and the second a method for obtaining the system natural frequencies.

The stability theory is based on the Liapunov direct method and makes use of modal analysis to represent elastic displacements. The novelty of the formulation lies in the fact that for the first time a nontrivial equilibrium is considered in conjunction with the Liapunov direct method for a stability analysis of spinning flexible bodies capable of large deformations.

The stability analysis can be divided into two major parts: the evaluation of the nontrivial equilibrium and the stability analysis itself.
When the body is capable of large deformations, nonlinear algebraic and differential equations must be solved for the rotational and elastic displacements, respectively, where these displacements define the equilibrium configurations of the system. Because the problem is one of stability about nontrivial equilibrium, it is necessary to expand the Liapunov function about that equilibrium. Assuming small displacements from equilibrium, the problem reduces to the evaluation of a Hessian matrix at the nontrivial equilibrium and testing the matrix for sign definiteness by means of Sylvester's criterion. It should be pointed out that the size of the Hessian matrix depends on the number of eigenfunctions used to represent the elastic displacements.

The method for obtaining the natural frequencies of the system makes use of the variational equations about the nontrivial equilibrium. Then the set of second-order differential equations is converted into a set of twice the number of first-order differential equations. The associated eigenvalue problem yields the system natural frequencies.

The two methods are quite general in scope, and can be used for testing stability and calculating the natural frequencies of a large variety of hybrid systems. As an application, the theory has been used to test the stability of the RAE/B satellite. First, the nonlinear equations have been solved for the nontrivial equilibrium configuration, and then this configuration has been used to evaluate the associated Hessian matrix. The satellite was found to be stable. Then one of the systems parameters has been varied to predict at which point the equilibrium becomes unstable. The results are in line with the expectations. In addition, the system natural frequencies for oscillation about the
deformed equilibrium were calculated. The parametric study used in conjunction with the Liapunov stability analysis was used to examine how the frequencies are affected. The study resulted in the same instability statement.
14. **References**


FIGURE 1 - GENERAL MATHEMATICAL MODEL
FIGURE 2 - ELASTIC DISPLACEMENTS

FIGURE 3 - ORBITAL AXES AND BODY AXES
FIGURE 4 - FORCES ON ELASTIC BOOM

FIGURE 5 - DEFORMATION OF ELASTIC BOOM
FIGURE 6 - RADIO ASTRONOMY EXPLORER - LUNAR (RAE/B) SATELLITE
FIGURE 7. NONTRIVIAL EQUILIBRIUM CONFIGURATION