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# Development of S-BAND ANTENNA INTERFACE DESIGN

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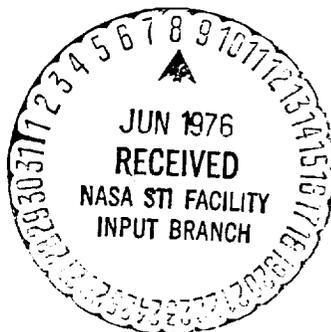
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## VOLUME I FINAL REPORT

MCDONNELL DOUGLAS ASTRONAUTICS COMPANY - EAST



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# Development of S-BAND ANTENNA INTERFACE DESIGN

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30 APRIL 1976

REPORT MDC E1478

## VOLUME I FINAL REPORT

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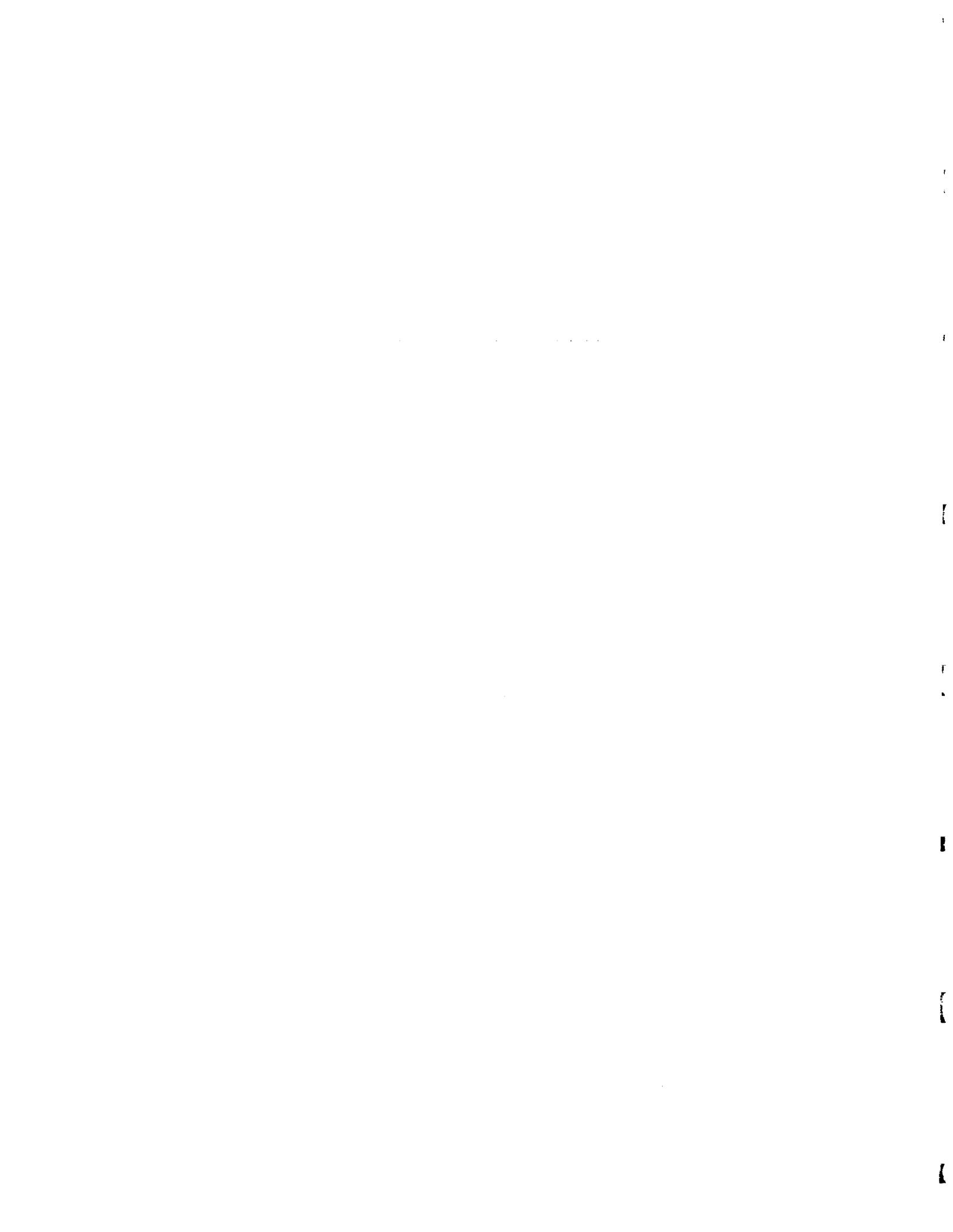
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**ABSTRACT**

The construction of an analytical thermal model of an S-band antenna in a typical Space Shuttle Orbiter installation is discussed. The selection and modeling of orbital and entry thermal environment inputs for the thermal analyses are discussed. The results of analyses for a variety of orbital thermal environments and entry initial conditions are given. Design and fabrication details of a thermal test fixture which physically approximates the Orbiter installation are discussed. The design and fabrication of two electrical test fixtures which electrically simulate the Orbiter surface shape and thermal protection system are discussed.

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**CONCLUSIONS**

The conclusions reached from an evaluation of the results obtained during this program are:

- (a) The antenna temperature extremes are within specification requirements for military aircraft antennas (-66 to 177°C (-85 to 350°F)).
- (b) Large temperature gradients do not exist in the antenna during any environmental condition because of the insulation afforded by the TPS.
- (c) An active thermal control plan is not required.
- (d) The rate of temperature change of the antenna due to orbital heat flux variations is slow.
- (e) The temperature of the interior interface (i.e., crew cabin wall) moderately affects the antenna temperatures.
- (f) Continuous antenna operation over extended periods raises the antenna temperatures approximately 11°C (20°F).
- (g) A thermal test fixture has been designed and fabricated which is independent of TPS thickness and can be used for both orbital and entry heating environments.
- (h) Electric test fixtures have been designed and fabricated which simulate the TPS and local shape for upper and lower Shuttle Orbiter body locations.

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**INTRODUCTION**

The purpose of this program was: (1) to construct a thermal model of an S-band antenna installed in a typical spacecraft structure, which is thermally protected by an external thermal protection system (TPS) and to analyze both orbital and entry thermal environments; (2) to design, fabricate and deliver a thermal test fixture which can be used to verify the thermal analyses by thermal vacuum tests; and (3) to design, fabricate and deliver two electrical test fixtures which simulate large and small radius spacecraft structures that are covered with external insulation for thermal protection during entry.

The Space Shuttle Orbiter is representative of current and future generation spacecraft designs and was, therefore, used as the baseline for the thermal analyses, the thermal test fixture and the electrical test fixtures required by this program. The Orbiter, unlike previous manned spacecraft which depended on ablating heat shields to dissipate entry heating, is covered with a thermal protection system (TPS) consisting principally of a low density, reusable, silica insulation bonded to the Orbiter structural skin. The insulation is locally sized to limit the maximum skin temperature. However, since the antenna aperture provides a surface discontinuity and a local heat sink, the antenna temperatures can differ from those on the surrounding skin surface. In terms of antenna performance, the thermal insulation is equivalent to covering the antenna and the surrounding ground plane with a dielectric sheet which affects the radiation patterns and impedance and the coupling to other antennas on the spacecraft.

An analytical thermal model of an S-band antenna mounted in the Space Shuttle Orbiter was constructed. The model includes a detailed representation of the antenna components, the Orbiter structure, the TPS internal insulation and heat sources.

The thermal model was used to determine the response of the antenna to a variety of orbital environments and to aerodynamic heating during entry from either hot or cold orbital conditions. The space irradiation was determined for spacecraft orbits in the Space Shuttle Reference Mission II timelines. Worst case hot and cold antenna temperatures were obtained for both operating and nonoperating modes of the antenna. The results of these analyses show that the temperatures, at any point on the antenna, remain between -63 and 128°C (-82 and 262°F) for all conditions considered.

A thermal test fixture was designed, fabricated and delivered. The test fixture design was based on the analytical thermal model to permit direct comparison of data from thermal vacuum tests with results obtained from that analytical model. Thermocouple locations on the tests fixture were selected to coincide with nodes in the analytical model.

Two electrical test fixtures were designed, fabricated and delivered. The test fixture shapes and simulated TPS configurations were based upon antenna locations on the upper and lower forward Orbiter body. The upper and lower Orbiter body locations represent large and small radius of curvature antenna ground planes, respectively.

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Mr. E. A. Kuhlman, Program Manager, was responsible for the overall technical direction of this program. Other members of the McDonnell Douglas Astronautics Company-East (MDAC-E) engineering staff who contributed to this program were L. H. Ebbesmeyer, E. A. Eiswirth, H. W. Kipp, G. D. Mitchell and C. D. Poore. Mr. Eiswirth provided the major effort for the design of the thermal model and performed the thermal analyses. Mr. R. B. Lewis, MCAIR Numerical Drafting Group, developed the frame contours of the simply curved surfaces for the electrical test fixtures. Test fixture fabrication was supervised by E. F. Disser and B. J. Myers of the McDonnell Aircraft Co. (MCAIR), Advanced Material Fabrication Facility. Messers. Disser and Myers also offered numerous suggestions which simplified the electrical and thermal test fixture designs.

Mr. H. D. Cubley of the Antenna System Section of the Electromagnetic Systems Branch, NASA Lyndon B. Johnson Space Center, Houston, Texas was the technical monitor for this program.

The units used for the physical quantities defined in this report are given in both the International System of Units (SI) and the U.S. customary units in the text. U.S. customary units are used in the drawings and in some of the graphs and tables.

This report is designated as Volume I in anticipation of additional work related to this contract which will be reported in a subsequent volume.

The task numbers used in the text correspond to the tasks given in the Statement of Work, Exhibit "A," of Contract NAS 9-14722, dated 1 July 1975.

## THERMAL ANALYSES

The Thermal Analyses Task (Task 2.3.1) consisted of developing an analytical thermal model of an S-band antenna (design specified by NASA-JSC) and selecting orbital and entry thermal environments based on the Space Shuttle Baseline Reference Mission II. The objectives were: (1) to determine the dynamic temperature response of the major structural features of the array antenna; (2) to determine the necessity for supplying power to maintain the antenna temperature; and (3) to prepare a thermal control plan for maintaining the antenna within acceptable thermal limits. This section discusses the analytical thermal model, the thermal environment and the thermal response of the antenna. The results of these analyses indicate that active thermal control is not required.

## Thermal Model

An antenna thermal model was constructed to evaluate the effects of orbital and entry thermal environments on an S-band antenna, mounted in a typical Space Shuttle Orbiter installation at both upper and lower body locations (figure 1). The external Orbiter wall consists of an aluminum skin-stringer structure covered with a reusable surface insulation (RSI), which combined with a strain isolation pad (SIP), serves as the thermal protection system (TPS). The RSI is bonded to the SIP and the SIP to the skin with RTV-560. Thermal properties of the antenna and TPS materials included in the thermal model are tabulated in Appendix A. A typical cross-section of TPS and structure is shown in figure 2. The thickness of the RSI is dependent upon the local heat input. The antenna is mounted in this structure with a supporting frame which holds the antenna aperture surface (typically a dielectric cover) flush with the skin surface. The TPS covers, and is bonded to, the antenna surface in the same way as it is to the surrounding skin. The RSI is not continuous but is cut into 15.2 x 15.2 cm (6.0 x 6.0 in.) tiles. The tile rows are offset 7.6 cm (3.0 in.) to minimize the gap heating. The gap width between tiles is nominally  $1.27 \pm .51$  mm ( $0.050 \pm 0.020$  in.). Heating at the bottom of the gaps is insignificant and is not considered in the thermal model. The antenna configuration and its relationship to the surrounding structure, internal insulation and heat sources greatly influence the antenna temperature.

Baseline antenna design. - The analytical model was based on the design shown in figure 3. The antenna consists of five circularly polarized radiating elements arranged in a symmetrical array. The elements are essentially open-ended ridged waveguides with orthogonal ridge pairs excited from a 90° hybrid at the base of each element. The elements are fed from a power division and phasing circuit board, which also serves as a mechanical base for holding the elements in place. The circuit board assembly is attached at three points to the sleeves surrounding each radiating element. The sleeves hold the elements in place, flush with the ground plane.

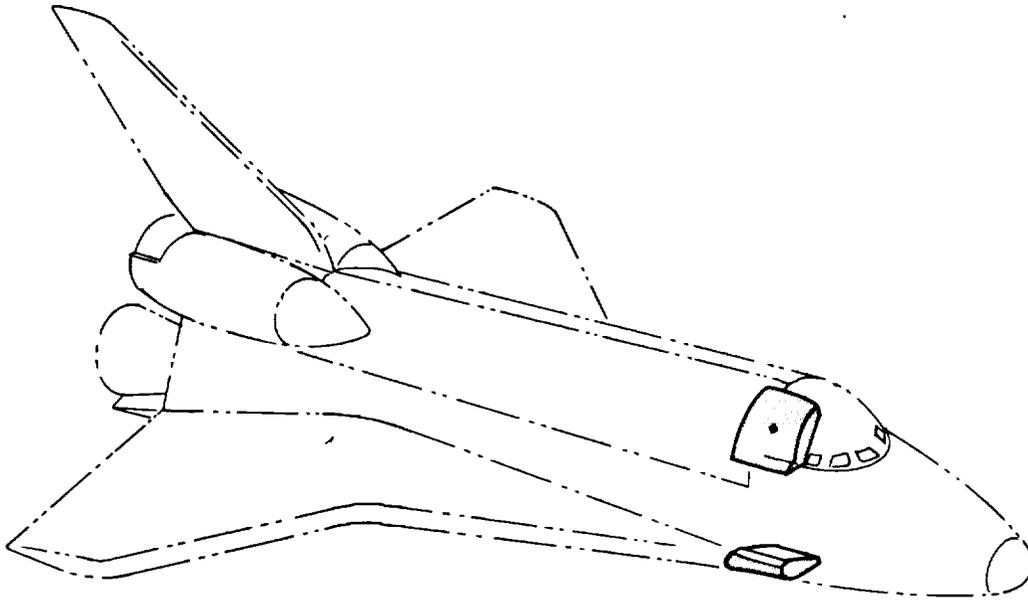
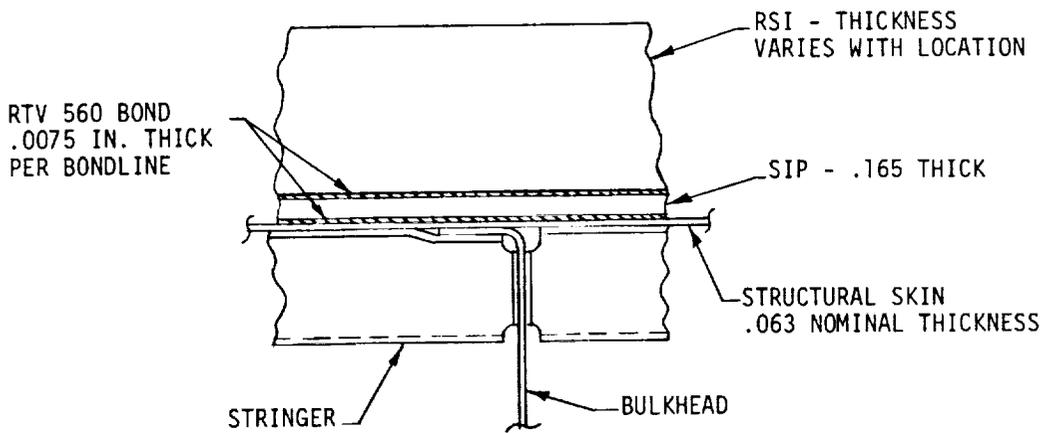


FIGURE 1 S-BAND ANTENNA LOCATIONS ON SPACE SHUTTLE ORBITER

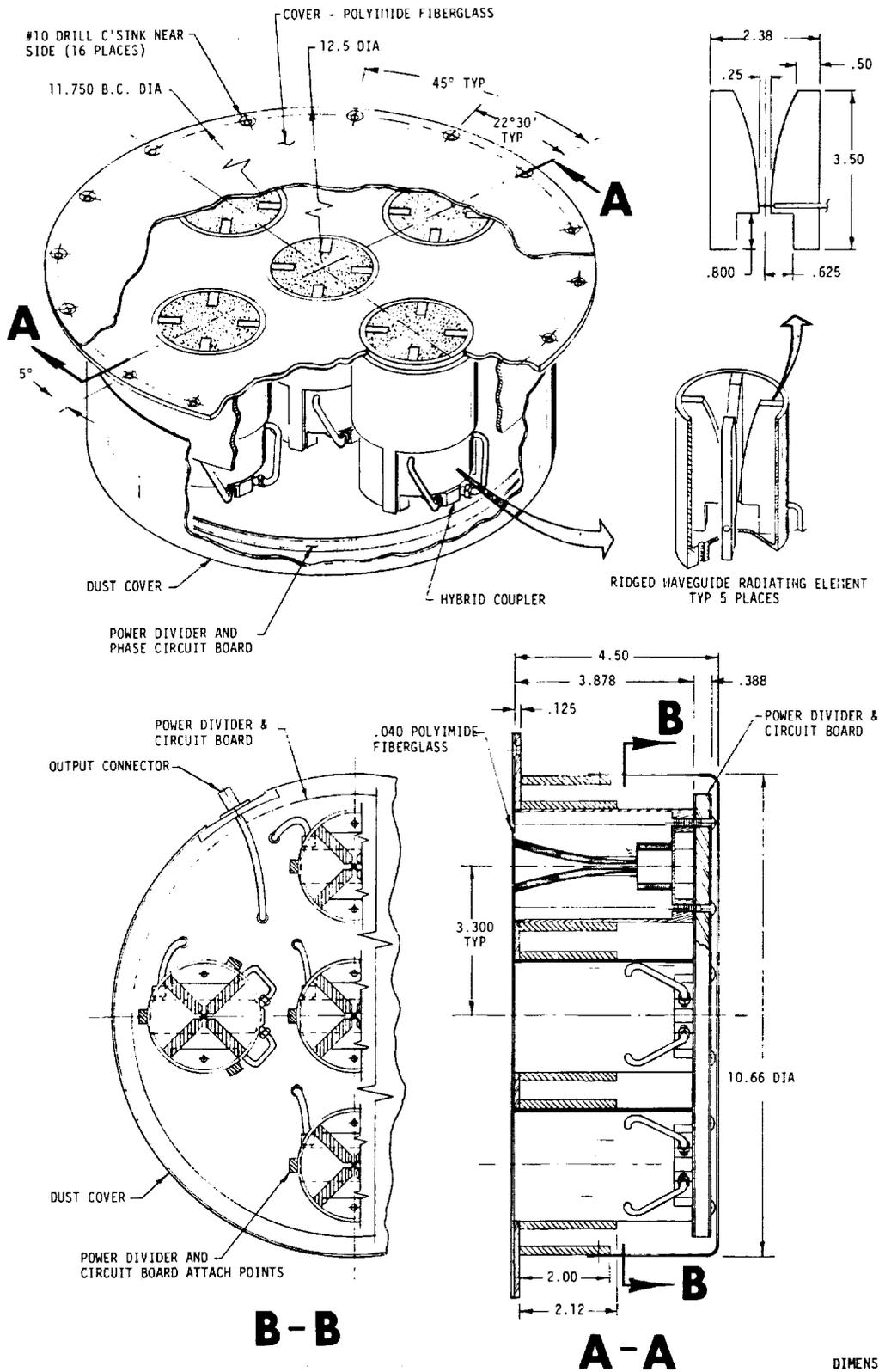


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FIGURE 2 TYPICAL TPS/STRUCTURE CONFIGURATION

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FIGURE 3 BASELINE S-BAND ANTENNA

Analytical thermal model. - A 324 node thermal model (ANTENA) of the S-band antenna and its surrounding structure and insulation was constructed using the MDAC-E thermal analysis program, HEATRAN. The model configuration, illustrated in figure 4, includes a 1/4 (90 degree) section of the antenna and a 61 cm (24 in.) square section of the surrounding skin-stringer structure and TPS. This size was required so that the edges of the model would be approximately adiabatic in the transverse direction. The distribution of the nodes of the model is as follows:

Skin and TPS	111 nodes
Antenna radiating elements	100 nodes
Circuit board, dust cover, antenna flange and plate	49 nodes
Internal insulation	12 nodes
Radiosity	52 nodes

The location of each of the thermal model nodes is tabulated in Appendix B. Figure 5 shows the arrangement of the model thermal nodes in the TPS, structure and internal insulation surrounding the antenna. Since only the antenna and the TPS bondline temperatures are of interest and since the transverse temperature gradients in the TPS are small with respect to the normal gradients, the curvature of the surface was neglected. The impressed heating rates were assumed to be constant over the model surface and the TPS thickness was assumed to be equal to that directly above the antenna, i.e., 1.04 cm (0.415 in.) and 4.44 cm (1.748 in.) for the upper and lower antenna locations, respectively. This is a reasonable approximation since the local TPS thickness is sized to limit the maximum skin temperature regardless of the local heating. It was necessary to model the skin and TPS in rectangular coordinates even though the antenna is rotationally symmetric because of the stringers attached to the vehicle skin. The conductance in the direction parallel to the stringers is almost double that along the perpendicular path. To accommodate this condition, the skin and the stringers were lumped together as nodes with different conductances in the parallel and perpendicular directions. The 111 nodes in the TPS and skin are arranged in seven layers of sixteen nodes each as shown in figure 5. The RSI includes five of the seven, equidistant nodes in the normal direction. The RTV bondlines and SIP were combined into one lumped node in the normal direction.

Figures 6 thru 9 illustrate the node locations within the antenna. Figure 6 is the top view; the cross-sectional views designated in figure 6 are shown in figures 7 and 8. Figure 9 is a detailed schematic of the nodes within one of the radiating elements. The nodal distribution is sufficiently detailed in the antenna section of the model to evaluate component interface temperatures and gradients. The forward radiating element contains thirty nodes and the associated hybrid coupler nine. The center radiating element is divided into eighteen nodes and its coupler contains six. The outboard radiating element has thirty nodes and its coupler has seven. The thermal model program is written so as to allow power to be dissipated in the element ridges to simulate heating due to the losses in the antenna during operation. All of the radiating element external nodes, the dust cover, mounting flange and circuit board are connected in the thermal model via a radiosity network which accounts for

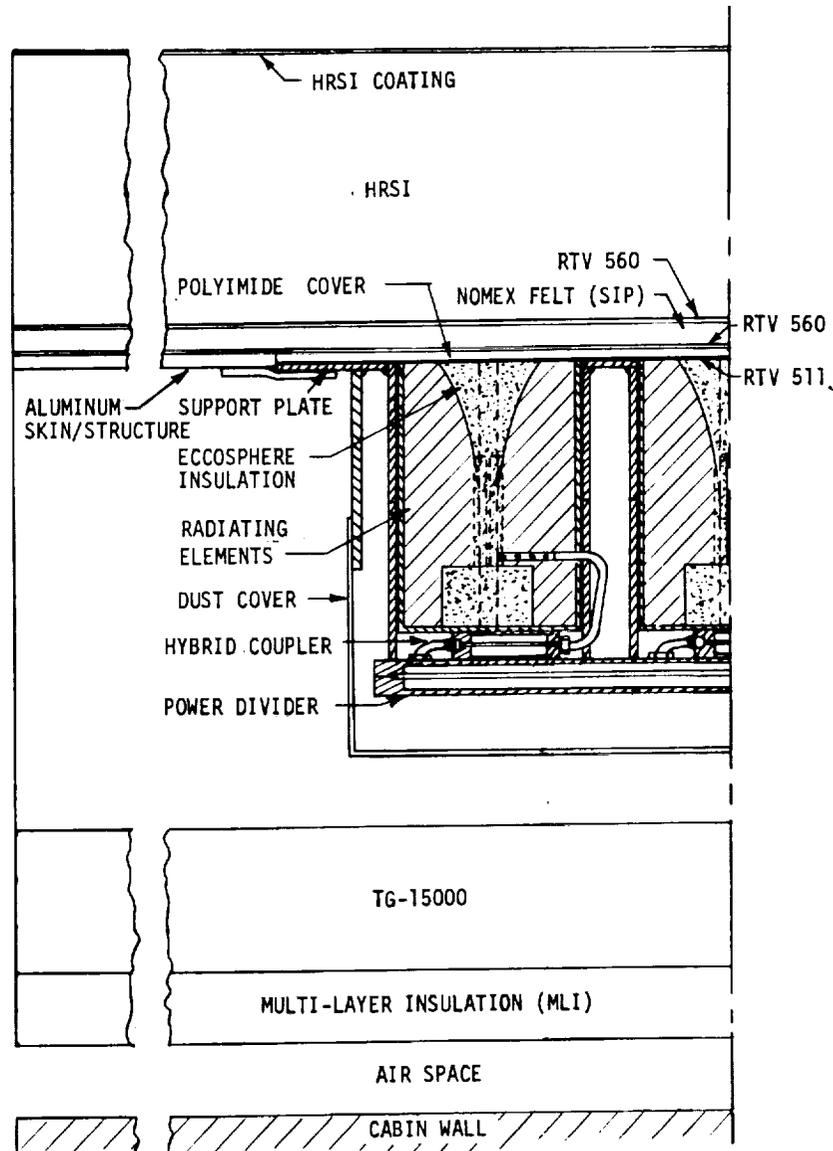


FIGURE 4 CROSS-SECTION OF S-BAND ANTENNA ANALYTICAL THERMAL MODEL

heating due to gray body radiation among components. Geometric view factors within the antenna were calculated using the MDAC-E thermal radiation analyzer program (RADQ).

The internal insulation (figure 4) of the thermal model, a 61 cm (24 in.) square section, contains only one node in the transverse direction. However, the insulation layers are divided into eleven nodes in the normal direction. This simplification causes no significant error since the cabin wall temperature is constant and the transverse temperature gradients are small. Radiosity connectors are used to link the antenna dust cover to the inside of the skin and to the TG-15000 insulation.

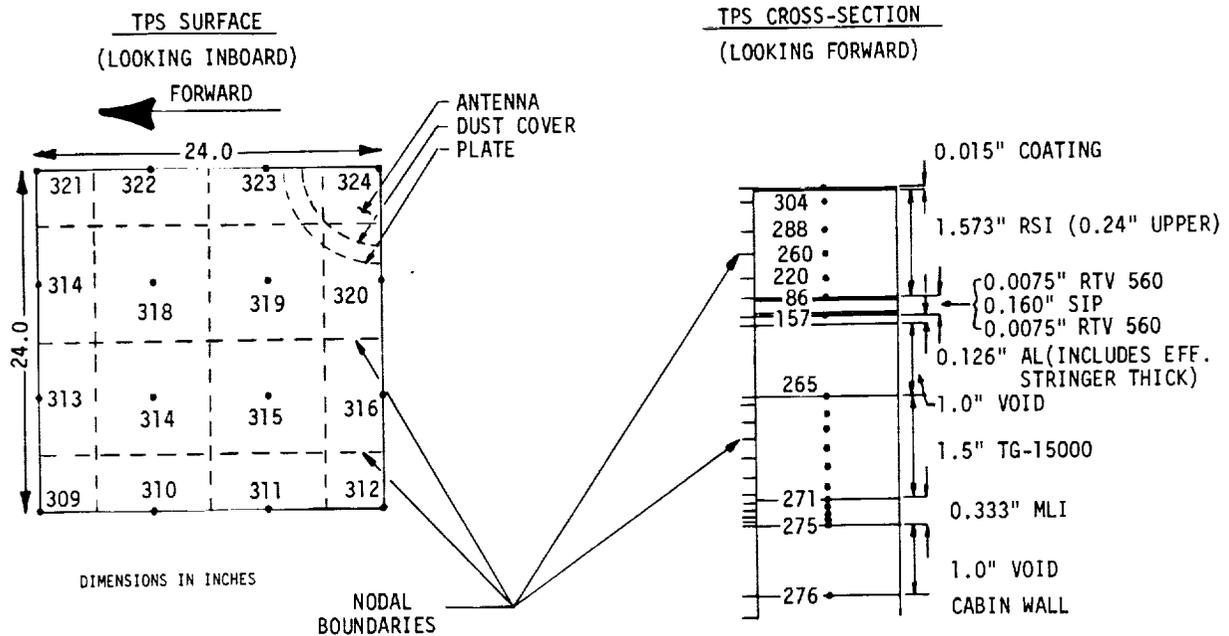


FIGURE 5 ANTENNA THERMAL MODEL NODAL DISTRIBUTION - TPS/STRUCTURE/INTERNAL INSULATION

The HEATRAN program uses the initial temperature level of each of the 324 nodes in the thermal model along with the boundary conditions, cabin wall temperature and TPS surface heat transfer rate history to determine the transient temperature response at each of the nodes. Orbital equilibrium temperatures are determined by computing the temperature changes over several orbits and extrapolating each node temperature history in an exponential manner to the equilibrium level. The matrix solution techniques used in these analyses were either Gaussian elimination or Gauss-Seidel iteration depending on the rate of change in the applied heat flux. A sample output listing is shown in Appendix C.

### Thermal Environment

The S-band antenna thermal environment consists of two parts, that associated with the external TPS surface and that at the interface within the vehicle. The external environment is composed of either irradiation from the Sun and Earth during orbital flight or aerodynamic heating during entry. The orbital environment was derived from an analysis of the Space Shuttle Baseline Reference Mission II timelines for Options 1, 2 and 3. The entry environment was based on entry surface temperatures provided by NASA. The internal environment is controlled by the capability of the antenna to accept or reject heat from the spacecraft interior.



VIEW B - B

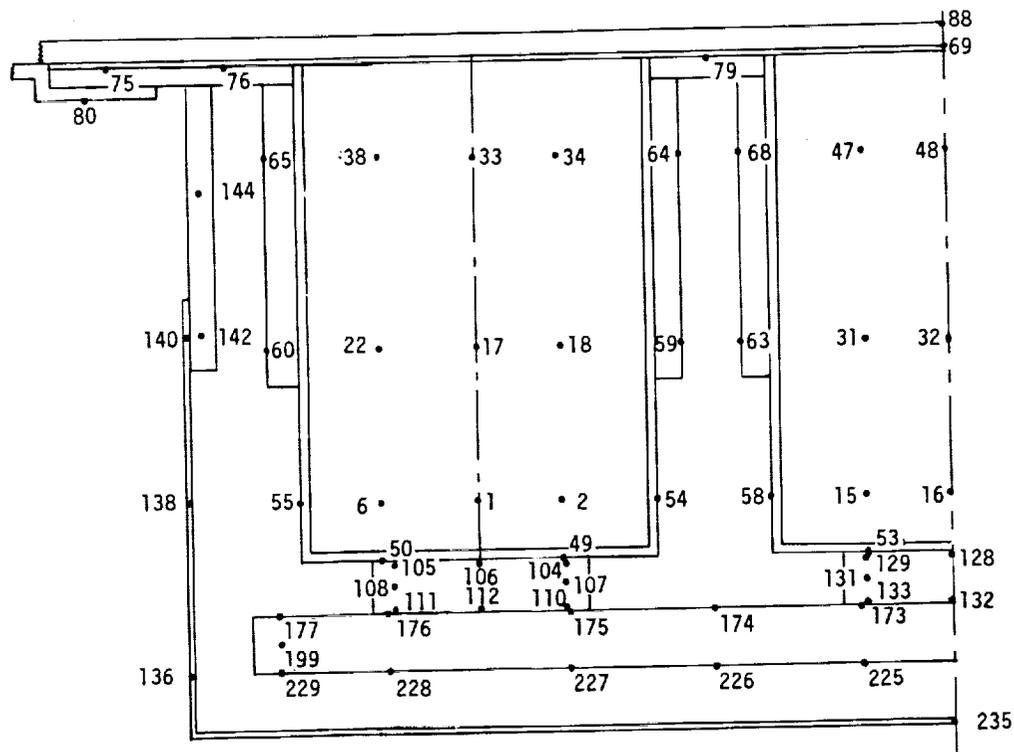


FIGURE 7 ANTENNA THERMAL MODEL NODAL DISTRIBUTION - ANTENNA CROSS-SECTION LOOKING OUTBOARD

at a specified surface is dependent on the spacecraft orbit and attitude because: (1) the irradiation is a function of the angle between the receiver and the radiation source and (2) part or all of a surface may be shaded by other surfaces or by the Earth.

The thermal analyses of the antennas at the respective locations were conducted for the extreme heat flux conditions, i.e., the maximum and minimum total absorbed heat flux. This approach yielded the upper and lower antenna temperature extremes. All other orbits would provide antenna temperatures within this range. In addition to the extreme cases, a spacecraft thermal conditioning case typical of all Mission II options was analyzed to provide typical initial antenna temperatures for entry. The incident and absorbed values of each of the thermal radiation fluxes were determined using the RADQ program. This program accounts for the orientation and optical properties of the receiving surface and also the extent and duration of shading during the orbit.

Since the principal portion of the total irradiation is direct solar, maximum and minimum heating conditions were approximated by rotating the spacecraft such that each antenna surface received sunlight over its entire area for all or none of the orbit. The percentage of a mean solar day that a

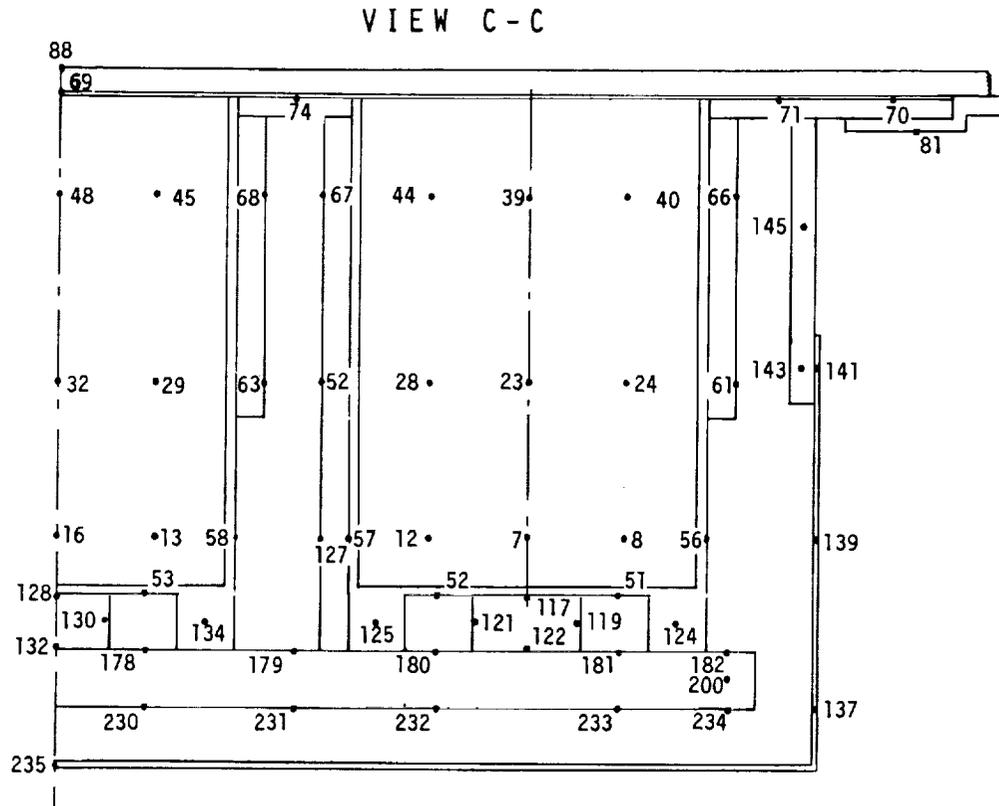


FIGURE 8 ANTENNA THERMAL MODEL NODAL DISTRIBUTION - ANTENNA CROSS-SECTION LOOKING AFT

vehicle in orbit sees the Sun is determined by the beta angle,  $\beta$ , and the altitude. This angle is defined as the angle between the orbit plane and the Earth/Sun line and depends on: (1) the orbit inclination angle,  $i$ , (2) the longitude of the ascending node,  $\Omega$ , and (3) the time of year. From figure 10 it can be shown that the beta angle reduces to the inclination angle when the solar declination angle,  $\delta$ , is zero and the ascending node angle,  $\Omega$ , is  $90.0^\circ$ , since the Sun line and the reference line then coincide. For small inclination angles, figure 10 shows part of the orbit path is shaded by the Earth. For a typical Space Shuttle Orbiter altitude of 500 km, beta angles greater than about  $68.5^\circ$  provide 100% of sunlight. However, as  $\beta$  increases the amount of thermal radiation received from the Earth decreases. Therefore, a high total flux condition is provided by an orbit in which the beta angle is near the minimum 100% Sun angle with the receiving surface normal to the Sun's rays. Conversely, a cold environment can be found in the same orbit by rotating the surface  $180^\circ$  such that no direct solar radiation is received.

Using the basic ground rule that any Orbiter solar inertial attitude may be maintained for up to 168 hours, a survey of the attitude and pointing timelines, contained in the Space Shuttle Baseline Reference Mission II - Options 1, 2, and 3, was conducted to find the extreme antenna environments. No appropriate maximum or minimum heating conditions were found because the frequent attitude changes in these typical missions did not allow adequate

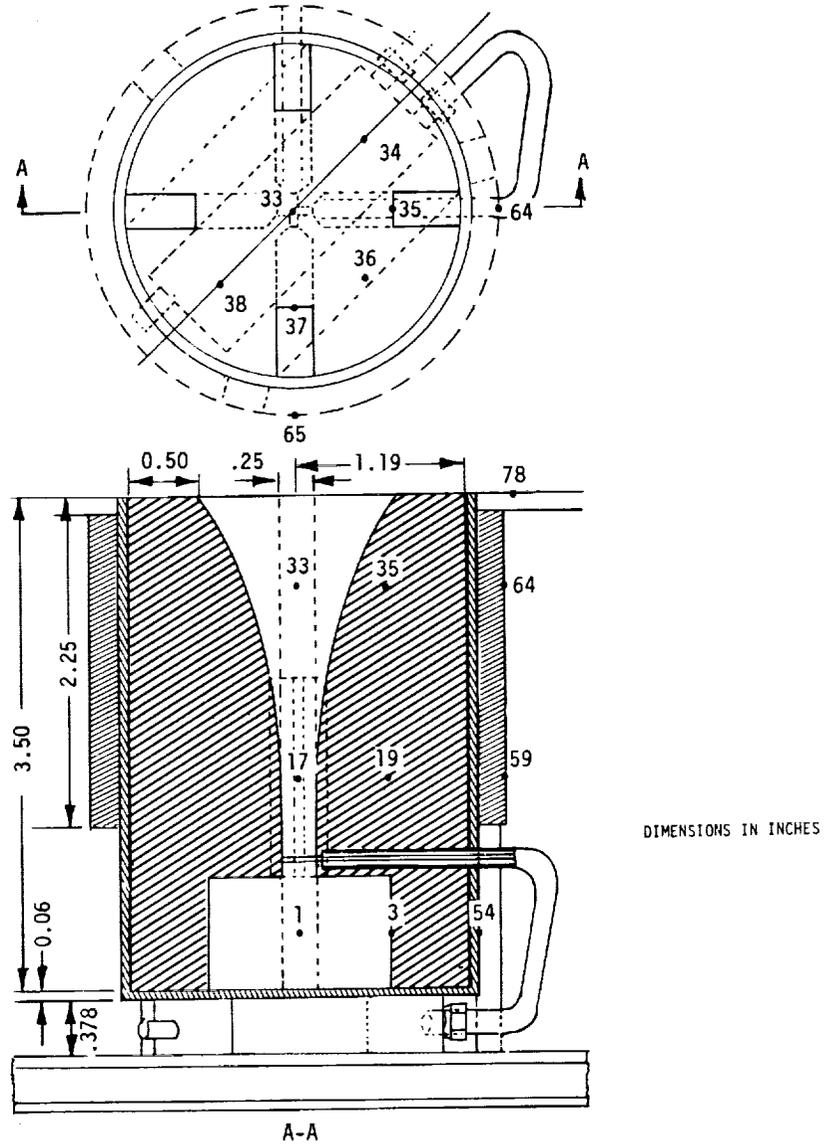
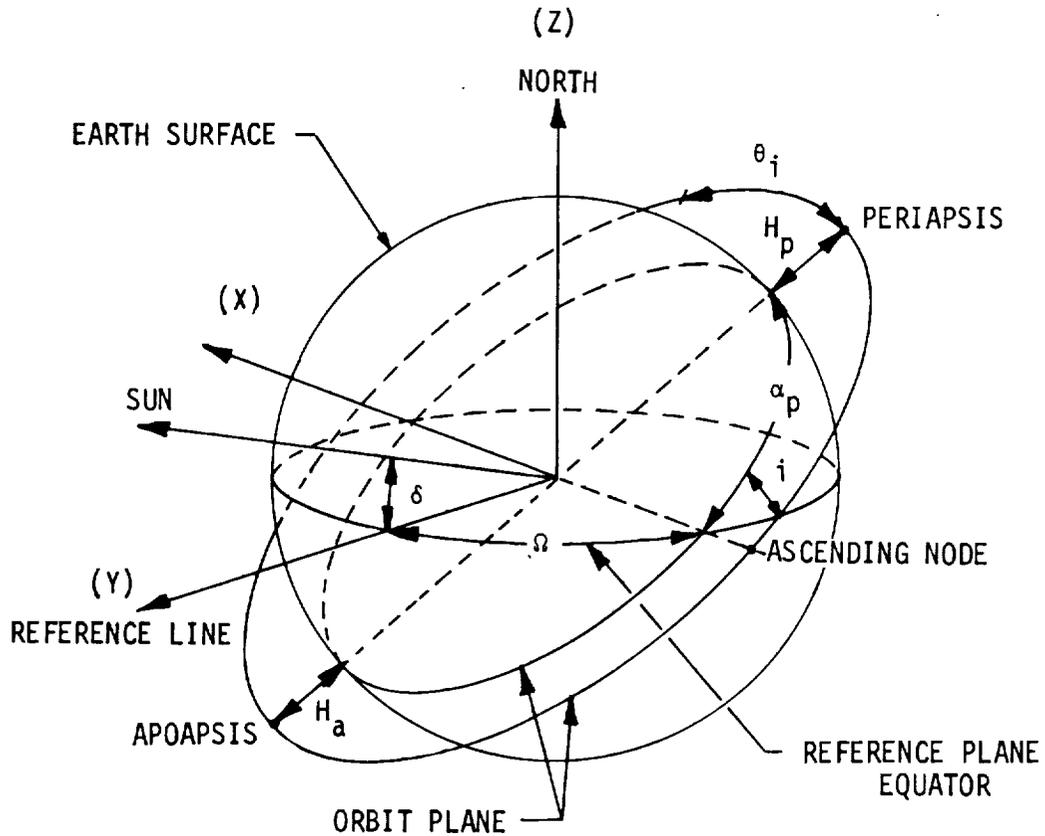


FIGURE 9 ANTENNA THERMAL MODEL - RADIATING ELEMENT NODAL DISTRIBUTION

time for complete heat soak to equilibrium. Therefore, an orbit, which met the above requirements, was chosen from Option 1. The vehicle was then rotated into the proper solar inertial attitudes to maximize or minimize the direct solar heating on the TPS surfaces over both the upper and lower antennas. In addition to these extreme cases, an orbit near the end of Option 1, in which thermal conditioning was occurring was also chosen for analysis. Table I shows the orbits and attitudes analyzed during this program. The orbital location parameters are shown in figure 10 while the local and body axis systems for the Sun oriented coordinate system are shown in figure 11.

Figure 12 shows the total absorbed heat flux histories for the hot and cold orbits (i.e., antenna heating maximized or minimized) while tables II

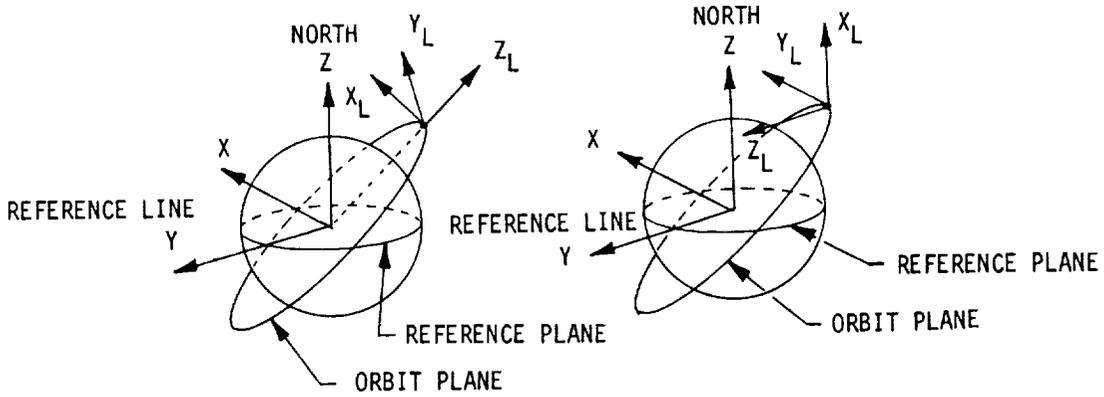


$$\beta = i \text{ when } \delta = 0.0 \text{ and } \Omega = 90.0$$

FIGURE 10 ELEMENTS OF ORBITAL MOTION

TABLE I  
SPACECRAFT ORBITS AND ATTITUDES ANALYZED

CONDITION	OPTION 1 TIME (SEC)	$\beta$ DEG	$\phi$ DEG	$\psi$ DEG	$\omega$ DEG	% TIME IN SUN	COMMENTS
HOT LOWER ANTENNA	192 000	75.2	0	0	27	100	3 AXIS SOLAR INERTIAL
COLD LOWER ANTENNA	192 000	75.2	0	0	127	100	3 AXIS SOLAR INERTIAL
HOT UPPER ANTENNA	192 000	75.2	0	0	127	100	3 AXIS SOLAR INERTIAL
COLD UPPER ANTENNA	192 000	75.2	91	-3.8	75.6	100	3 AXIS SOLAR INERTIAL
THERMAL CONDITIONING	537 660	72.6	NA	0	NA	100	$\dot{\phi} = 0.5$ DEGREE/SECOND $\dot{\omega} = 0.5$ DEGREE/SECOND



EARTH ORIENTED

- $X_L$  = ORBITAL PLANE COMPONENT IN DIRECTION OF MOTION
- $Y_L$  = PERPENDICULAR TO ORBIT PLANE (RIGHT-HANDED)
- $Z_L$  = LOCAL ZENITH (AWAY FROM EARTH)

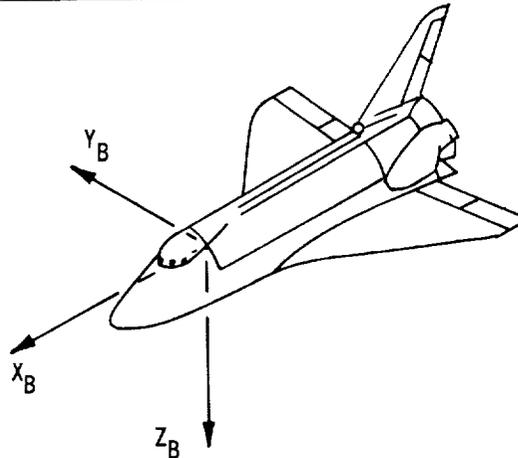
SUN ORIENTED

- $X_L$  = PARALLEL TO NORTH POLE OF REFERENCE PLANE
- $Y_L$  = FORMS RIGHT-HANDED SET
- $Z_L$  = PARALLEL TO REFERENCE LINE

A. ORBIT LOCAL COORDINATE SYSTEM

ROTATE LOCAL COORDINATE SYSTEM INTO VEHICLE BODY AXIS SYSTEM.

- $\phi$  : ROTATE ABOUT  $Z_L$ ,  $Y_L \rightarrow X_L$  IS POSITIVE
- $\psi$  : ROTATE ABOUT  $Y_L$ ,  $X_L \rightarrow Z_L$  IS POSITIVE
- $\omega$  : ROTATE ABOUT  $X_L$ ,  $Y_L \rightarrow Z_L$  IS POSITIVE



B. SPACECRAFT BODY AXIS SYSTEM

FIGURE 11 ELEMENTS OF SPACECRAFT ORBITAL ATTITUDES

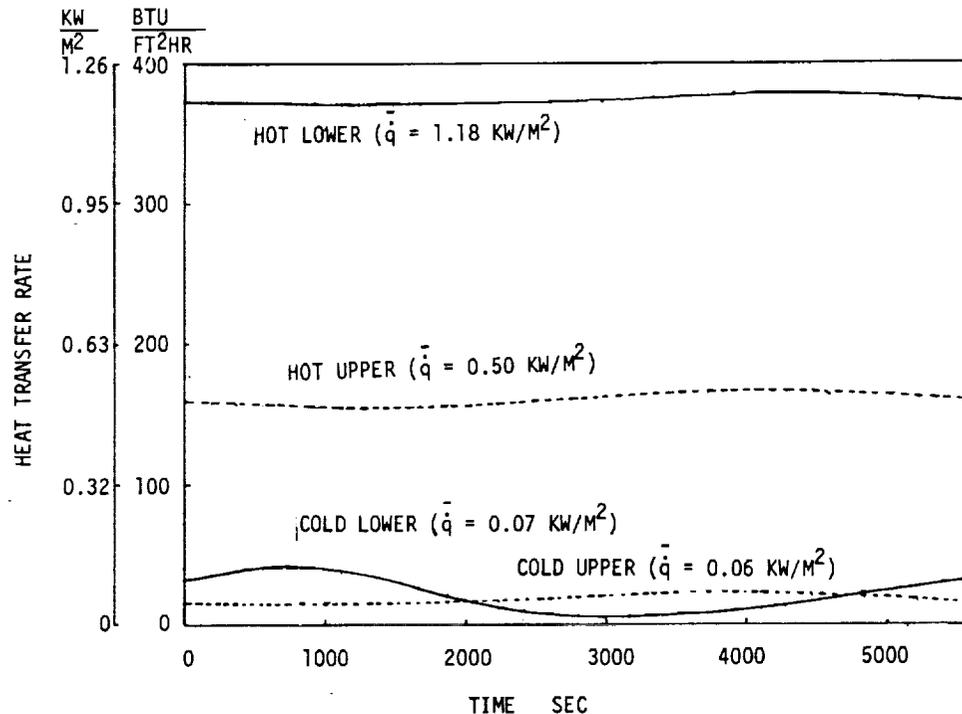


FIGURE 12 ANTENNA MAX/MIN ORBITAL ABSORBED HEATING HISTORIES

and III indicate the individual irradiation at each antenna location. The absorbed flux differs from the irradiation due to the absorptivity of the TPS surface coating. The total hemispherical emissivity of the TPS coating is 0.8. However, the solar absorptivity,  $\alpha_s$ , of the lower antenna TPS coating is 0.8 while that of the upper is 0.32. Therefore, during the hot orbits when the principal flux is solar, a much higher percentage of the irradiation is absorbed at the lower antenna location than at the upper. For the cold orbits the solar flux at the antenna is composed only of Earth reflected radiation and its magnitude is comparable to that of the Earth emitted radiation.

Figures 13 and 14 illustrate the total absorbed flux during thermal conditioning. In contrast with the hot and cold environments, the heat rates shown in these figures are highly variable. This is due to the rotation of the spacecraft which results in the antenna surfaces facing the Sun periodically even though the entire orbit is in sunlight as shown in table I.

Exterior boundary condition during entry. - During entry the principal mode of heat transfer to the TPS surface over the antenna is convection due to aerodynamic heating. Figures 15 and 16 show the entry TPS surface and bondline temperatures for the upper and lower locations specified for this program by NASA. (Note that the TPS bondline is between the SIP and skin. Later reference will be made to an RSI bondline which is between the RSI and the SIP.) Figure 17 indicates the corresponding TPS pressures during the deorbit trajectory. These temperatures were determined by applying a local heating rate to a one-dimensional model of the vehicle TPS and skin, as shown on

TABLE II  
LOWER ANTENNA MAX/MIN INCIDENT HEAT FLUX

HOT LOWER ANTENNA (+Y); ( $\phi$ ,  $\psi$ ,  $\omega$ ) = (0.0, 0.0, 27.0)

TIME	SOLAR	ALBEDO	PLANET	SUMMED FLUX
AVERAGES - - -	.440091	.367775	.219544	.46654E+03
.192000E+06	.440091	.350999	.209994	.46541E+03
.192190E+06	.440091	.516181	.192277	.46525E+03
.192370E+06	.440091	.624011	.176932	.46484E+03
.192560E+06	.440091	.753388	.162555	.46422E+03
.192750E+06	.440091	.756711	.150499	.46353E+03
.192930E+06	.440091	.785011	.141122	.46287E+03
.193120E+06	.440091	.797711	.134711	.46236E+03
.193310E+06	.440091	.802222	.131722	.46210E+03
.193500E+06	.440091	.796922	.132253	.46221E+03
.193680E+06	.440091	.780788	.137799	.46258E+03
.193870E+06	.440091	.745900	.144477	.46320E+03
.194050E+06	.440091	.688552	.155533	.46390E+03
.194240E+06	.440091	.591155	.168555	.46462E+03
.194420E+06	.440091	.459951	.183384	.46521E+03
.194610E+06	.440091	.299077	.200000	.46555E+03
.194800E+06	.440091	.133355	.218299	.46565E+03
.194990E+06	.440091	.060499	.238888	.46613E+03
.195170E+06	.440091	.031688	.258745	.46673E+03
.195360E+06	.440091	.048277	.276444	.46734E+03
.195550E+06	.440091	.082000	.291555	.46791E+03
.195730E+06	.440091	.117555	.303111	.46712E+03
.195920E+06	.440091	.155677	.310666	.46720E+03
.196100E+06	.440091	.175555	.313888	.46722E+03
.196280E+06	.440091	.186644	.312999	.46722E+03
.196470E+06	.440091	.196555	.307888	.46717E+03
.196650E+06	.440091	.201999	.299844	.46708E+03
.196840E+06	.440091	.213344	.285333	.46695E+03
.197020E+06	.440091	.224388	.268922	.46681E+03
.197210E+06	.440091	.227011	.250333	.46667E+03
.197400E+06	.440091	.182299	.229977	.46657E+03
.197590E+06	.440091	.350999	.209994	.46554E+03

COLD LOWER ANTENNA (-Y); ( $\phi$ ,  $\psi$ ,  $\omega$ ) = (0.0, 0.0, 127.0)

TIME	SOLAR	ALBEDO	PLANET	SUMMED FLUX
AVERAGES - - -	0.	.27455E+01	.232263E+02	.26012E+02
.192000E+06	0.	.188444E+00	.400000E+02	.40189E+02
.192190E+06	0.	.183224E+00	.415880E+02	.43413E+02
.192370E+06	0.	.466522E+01	.421770E+02	.46835E+02
.192560E+06	0.	.760111E+01	.418550E+02	.49451E+02
.192750E+06	0.	.922600E+01	.405880E+02	.50506E+02
.192930E+06	0.	.112199E+02	.383299E+02	.49548E+02
.193120E+06	0.	.113322E+02	.351999E+02	.46532E+02
.193310E+06	0.	.115412E+02	.315220E+02	.41934E+02
.193500E+06	0.	.890577E+01	.279557E+02	.36866E+02
.193680E+06	0.	.699676E+01	.243311E+02	.31288E+02
.193870E+06	0.	.487751E+01	.206920E+02	.25356E+02
.194050E+06	0.	.287744E+01	.171333E+02	.20011E+02
.194240E+06	0.	.127655E+01	.139108E+02	.15194E+02
.194420E+06	0.	.311900E+00	.115440E+02	.11860E+02
.194610E+06	0.	.46703E-02	.987280E+01	.98777E+01
.194800E+06	0.	0.	.851495E+01	.85145E+01
.194990E+06	0.	0.	.763533E+01	.76353E+01
.195170E+06	0.	0.	.724177E+01	.72417E+01
.195360E+06	0.	0.	.745374E+01	.74537E+01
.195550E+06	0.	0.	.822264E+01	.82226E+01
.195730E+06	0.	0.	.943233E+01	.94323E+01
.195920E+06	0.	0.	.110133E+02	.11013E+02
.196100E+06	0.	0.	.131233E+02	.13123E+02
.196280E+06	0.	0.	.161444E+02	.16144E+02
.196470E+06	0.	0.	.195999E+02	.19599E+02
.196650E+06	0.	0.	.232278E+02	.23227E+02
.196840E+06	0.	0.	.269111E+02	.26911E+02
.197020E+06	0.	0.	.304666E+02	.30466E+02
.197210E+06	0.	0.	.341433E+02	.34143E+02
.197400E+06	0.	0.	.375166E+02	.37516E+02
.197590E+06	0.	.18844E+00	.400001E+02	.40189E+02

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TABLE III

UPPER ANTENNA MAX/MIN INCIDENT HEAT FLUX

HOT UPPER ANTENNA (+Y); ( $\phi$ ,  $\psi$ ,  $\omega$ ) = (0.0, 0.0, 127.0)

TIME	SOLAR	ALBEDO	PLANET	SUMMED FLUX
AVERAGES----	.44091E+03	.36775E+01	.21954E+02	.46654E+03
.19200E+06	.44091E+03	.35116E+01	.21012E+02	.46543E+03
.19219E+06	.44091E+03	.50650E+01	.19293E+02	.46527E+03
.19237E+06	.44091E+03	.62444E+01	.17709E+02	.46486E+03
.19256E+06	.44091E+03	.70257E+01	.16226E+02	.46423E+03
.19275E+06	.44091E+03	.75717E+01	.15059E+02	.46354E+03
.19293E+06	.44091E+03	.78540E+01	.14120E+02	.46288E+03
.19312E+06	.44091E+03	.79796E+01	.13476E+02	.46237E+03
.19331E+06	.44091E+03	.80209E+01	.13175E+02	.46211E+03
.19349E+06	.44091E+03	.80196E+01	.13250E+02	.46218E+03
.19368E+06	.44091E+03	.79661E+01	.13696E+02	.46257E+03
.19387E+06	.44091E+03	.78036E+01	.14446E+02	.46318E+03
.19405E+06	.44091E+03	.74542E+01	.15520E+02	.46388E+03
.19424E+06	.44091E+03	.68484E+01	.16833E+02	.46460E+03
.19442E+06	.44091E+03	.59115E+01	.18366E+02	.46519E+03
.19461E+06	.44091E+03	.45924E+01	.19988E+02	.46549E+03
.19480E+06	.44091E+03	.29065E+01	.21810E+02	.46563E+03
.19498E+06	.44091E+03	.13352E+01	.23866E+02	.46611E+03
.19517E+06	.44091E+03	.56040E+00	.25859E+02	.46733E+03
.19535E+06	.44091E+03	.23106E+00	.27623E+02	.46877E+03
.19554E+06	.44091E+03	.94821E-01	.29144E+02	.47014E+03
.19573E+06	.44091E+03	.41818E-01	.30310E+02	.47126E+03
.19591E+06	.44091E+03	.22566E-01	.31006E+02	.47199E+03
.19610E+06	.44091E+03	.17525E-01	.31337E+02	.47231E+03
.19628E+06	.44091E+03	.18694E-01	.31229E+02	.47223E+03
.19647E+06	.44091E+03	.26507E-01	.30789E+02	.47173E+03
.19666E+06	.44091E+03	.59196E-01	.29989E+02	.47083E+03
.19684E+06	.44091E+03	.13545E+00	.28853E+02	.46955E+03
.19703E+06	.44091E+03	.32442E+00	.26600E+02	.46814E+03
.19722E+06	.44091E+03	.77083E+00	.23954E+02	.46674E+03
.19740E+06	.44091E+03	.18226E+01	.22497E+02	.46573E+03
.19759E+06	.44091E+03	.35116E+01	.21012E+02	.46543E+03

COLD UPPER ANTENNA (-Y); ( $\phi$ ,  $\psi$ ,  $\omega$ ) = (9.10, -3.8, 75.6)

TIME	SOLAR	ALBEDO	PLANET	SUMMED FLUX
AVERAGES----	0.	.13590E+01	.21899E+02	.23258E+02
.19200E+06	0.	.25184E-01	.19601E+02	.19626E+02
.19219E+06	0.	.36889E+00	.18694E+02	.19063E+02
.19237E+06	0.	.11860E+01	.17916E+02	.19103E+02
.19256E+06	0.	.21491E+01	.17339E+02	.19488E+02
.19275E+06	0.	.30467E+01	.16948E+02	.19993E+02
.19293E+06	0.	.37824E+01	.16754E+02	.20543E+02
.19312E+06	0.	.43384E+01	.16762E+02	.21101E+02
.19331E+06	0.	.46785E+01	.16973E+02	.21651E+02
.19349E+06	0.	.47856E+01	.17384E+02	.22170E+02
.19368E+06	0.	.46334E+01	.17988E+02	.22621E+02
.19387E+06	0.	.41991E+01	.18764E+02	.22963E+02
.19405E+06	0.	.34607E+01	.19689E+02	.23149E+02
.19424E+06	0.	.24469E+01	.20715E+02	.23316E+02
.19442E+06	0.	.12857E+01	.21806E+02	.23309E+02
.19461E+06	0.	.35420E+00	.22914E+02	.23226E+02
.19480E+06	0.	.17469E-03	.24012E+02	.24013E+02
.19498E+06	0.	0.	.25024E+02	.25024E+02
.19517E+06	0.	0.	.25888E+02	.25888E+02
.19535E+06	0.	0.	.26600E+02	.26600E+02
.19554E+06	0.	0.	.27100E+02	.27100E+02
.19573E+06	0.	0.	.27351E+02	.27351E+02
.19591E+06	0.	0.	.27340E+02	.27340E+02
.19610E+06	0.	0.	.27706E+02	.27706E+02
.19628E+06	0.	0.	.26557E+02	.26557E+02
.19647E+06	0.	0.	.25838E+02	.25838E+02
.19666E+06	0.	0.	.24948E+02	.24948E+02
.19684E+06	0.	0.	.23935E+02	.23935E+02
.19703E+06	0.	0.	.22839E+02	.22839E+02
.19722E+06	0.	0.	.21716E+02	.21716E+02
.19740E+06	0.	0.	.20613E+02	.20613E+02
.19759E+06	0.	.25184E-01	.19601E+02	.19626E+02

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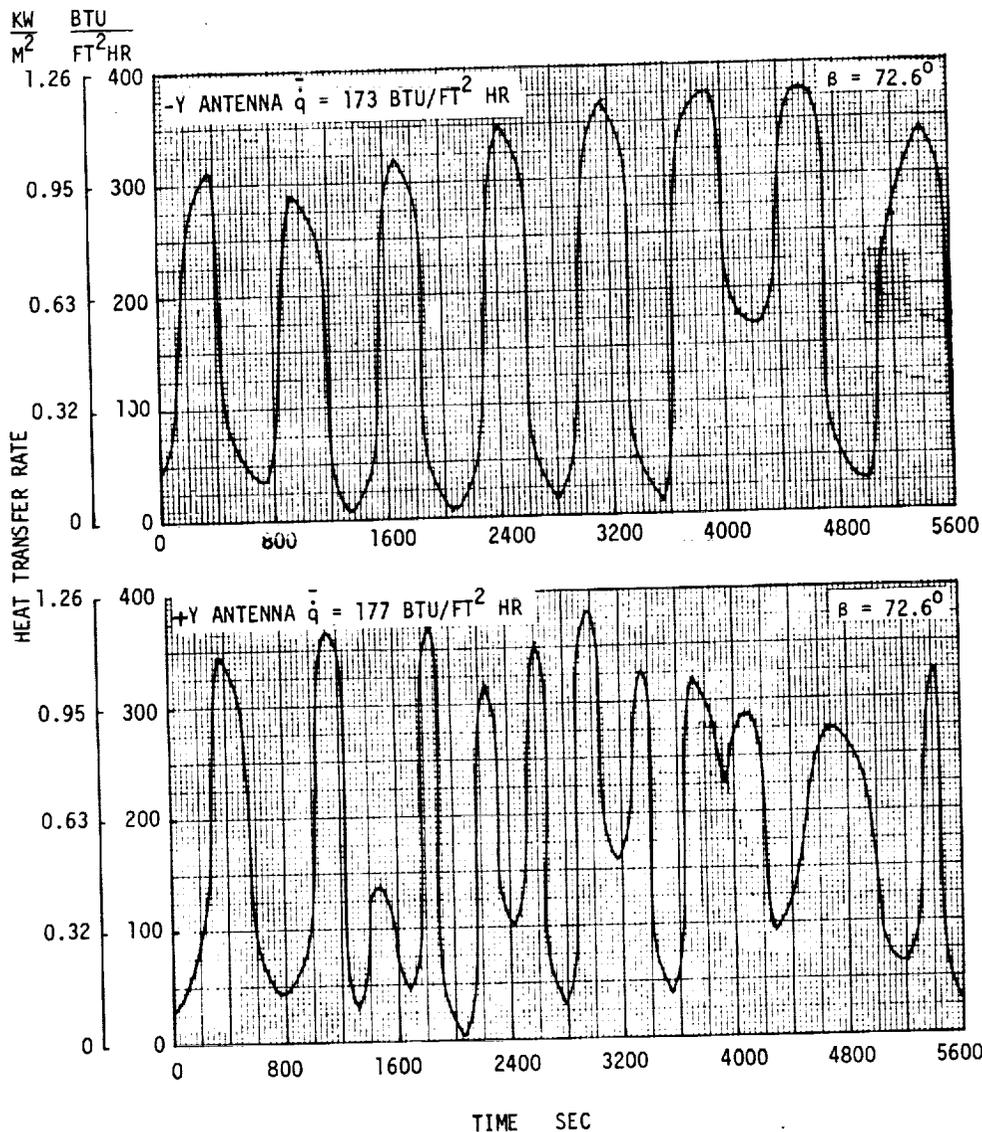


FIGURE 13 LOWER ANTENNA TOTAL ABSORBED HEATING DURING THERMAL CONDITIONING

figures 15 and 16 for a specified RSI thickness such that a maximum bondline temperature of 177°C (350°F) was attained. This type of analysis neglects heat leaks to the skin structural support and the effects of the interface with the vehicle interior. Consequently, the calculated bondline temperatures are higher than would be actually realized in the spacecraft.

The temperatures given on these curves correspond to those at locations where the RSI thicknesses are 8.23 cm (3.24 in.) and 0.84 cm (0.33 in.) for the lower and upper surfaces, respectively. The actual RSI thicknesses above the lower and upper array antennas are 3.99 cm (1.57 in.) and 0.61 cm (0.24 in.), respectively. These values correspond to TPS thicknesses of 4.45 cm (1.75 in.) and 1.07 cm (0.42 in.). Because of the differences in

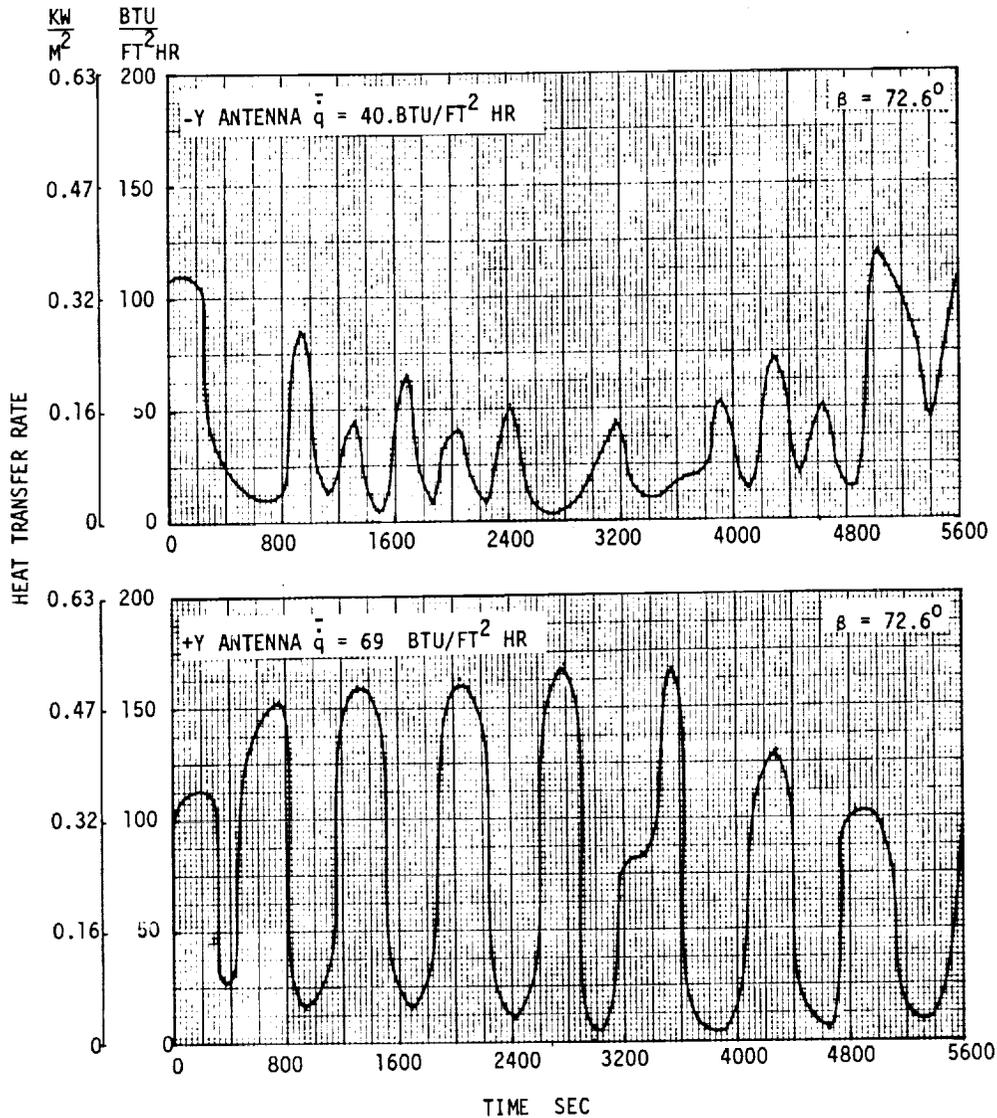


FIGURE 14 UPPER ANTENNA TOTAL ABSORBED HEATING DURING THERMAL CONDITIONING

insulation thickness use of the surface temperatures shown in figures 15 and 16 would yield bondline temperatures considerably in excess of 177°C (350°F). Therefore, it was necessary to scale the local heat transfer rates.

A one dimensional model, similar to that discussed above, was constructed and the surface temperatures shown in figures 15 and 16 were input into an inverse heat transfer program using the 8.23 cm (3.24 in.) and 0.84 cm (0.33 in.) RSI thicknesses. The resultant bondline temperature histories compared very well with those specified by NASA indicating that the NASA and MDAC-E models were equivalent. This program also supplies the convective heating rates corresponding to the impressed surface temperatures.

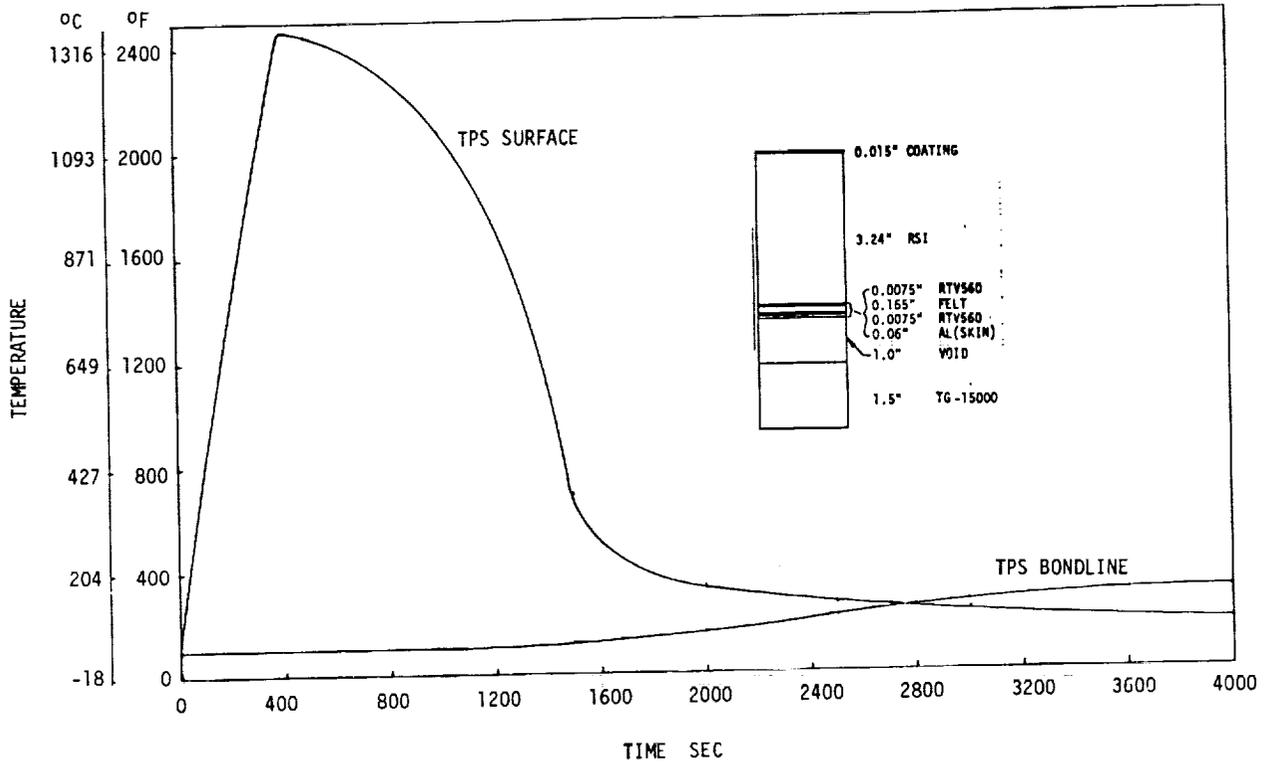


FIGURE 15 LOWER ANTENNA REGION TPS TEMPERATURES DURING ENTRY

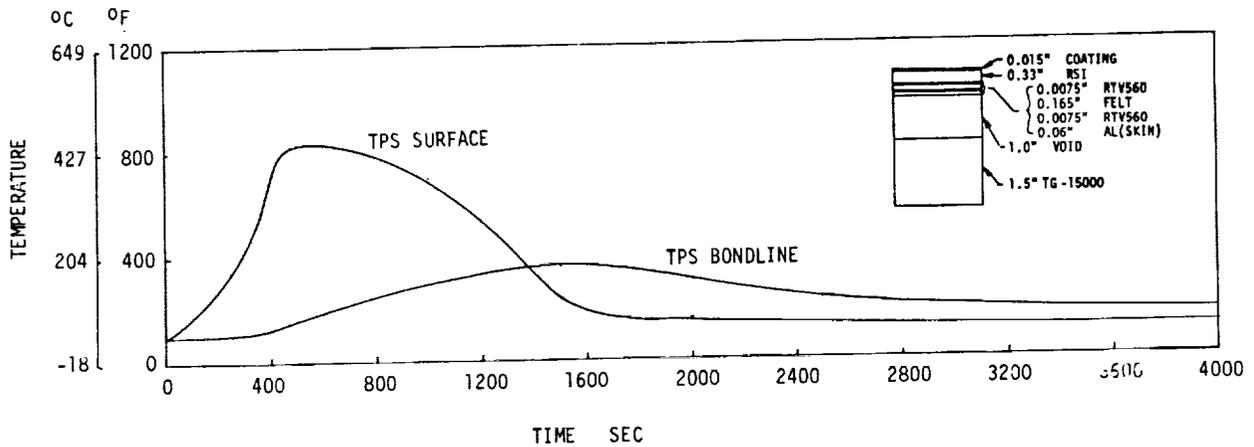


FIGURE 16 UPPER ANTENNA REGION TPS TEMPERATURES DURING ENTRY

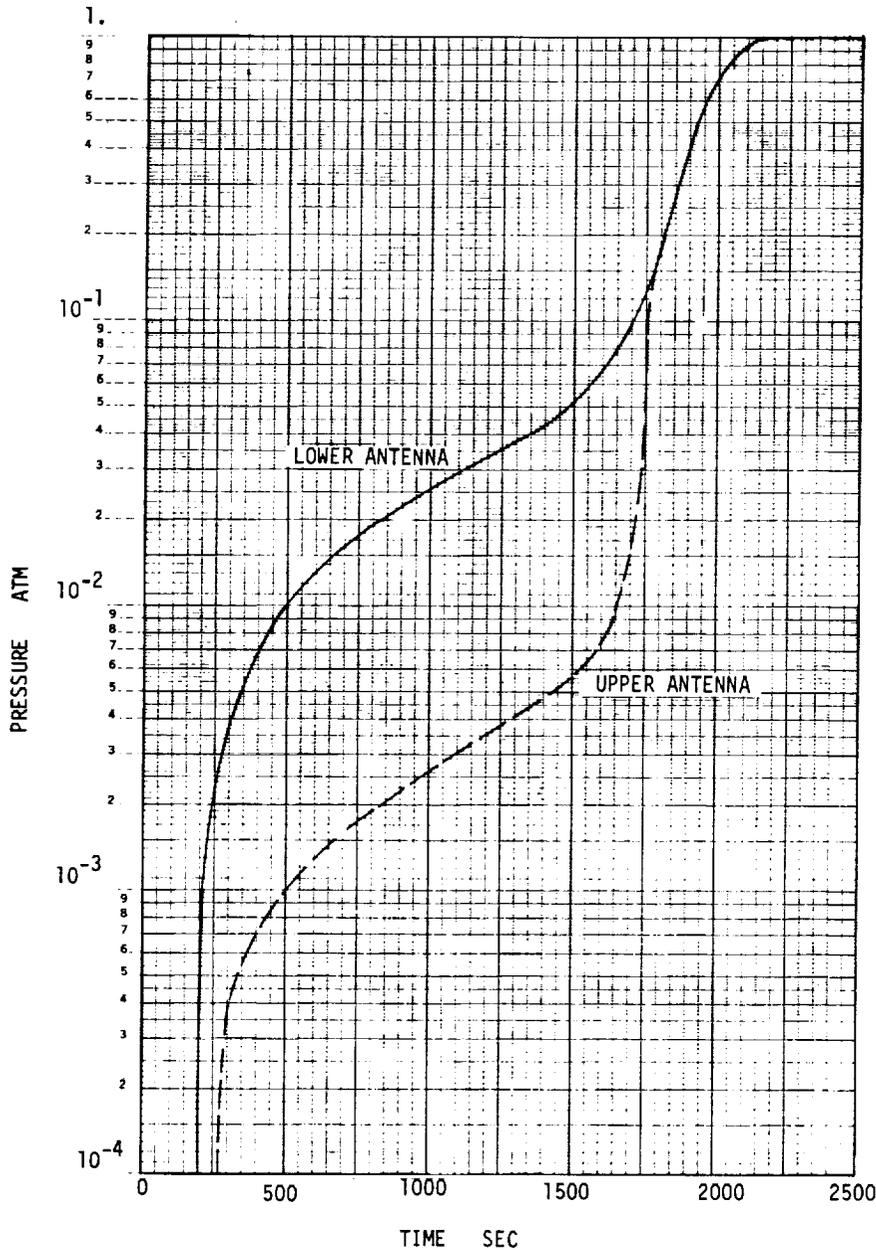


FIGURE 17 ANTENNA TPS SURFACE PRESSURE DURING ENTRY

The maximum heat fluxes for the upper and lower arrays were 13.13 and 316.13 kW/m<sup>2</sup> (4160 and 100 200 BTU/ft<sup>2</sup> hr), respectively during the trajectory. The RSI thicknesses were then changed to the actual values and the one dimensional model was run using a parametric application of a constant scale factor on the heating histories. This analysis indicated that heat transfer multipliers of 0.86 and 0.25 were required to maintain maximum TPS bondline temperatures at 177°C (350°F) for the upper and lower antenna locations, respectively. The calculated heating histories are shown in figure 18 and were used for all of the entry thermal analyses during this program.

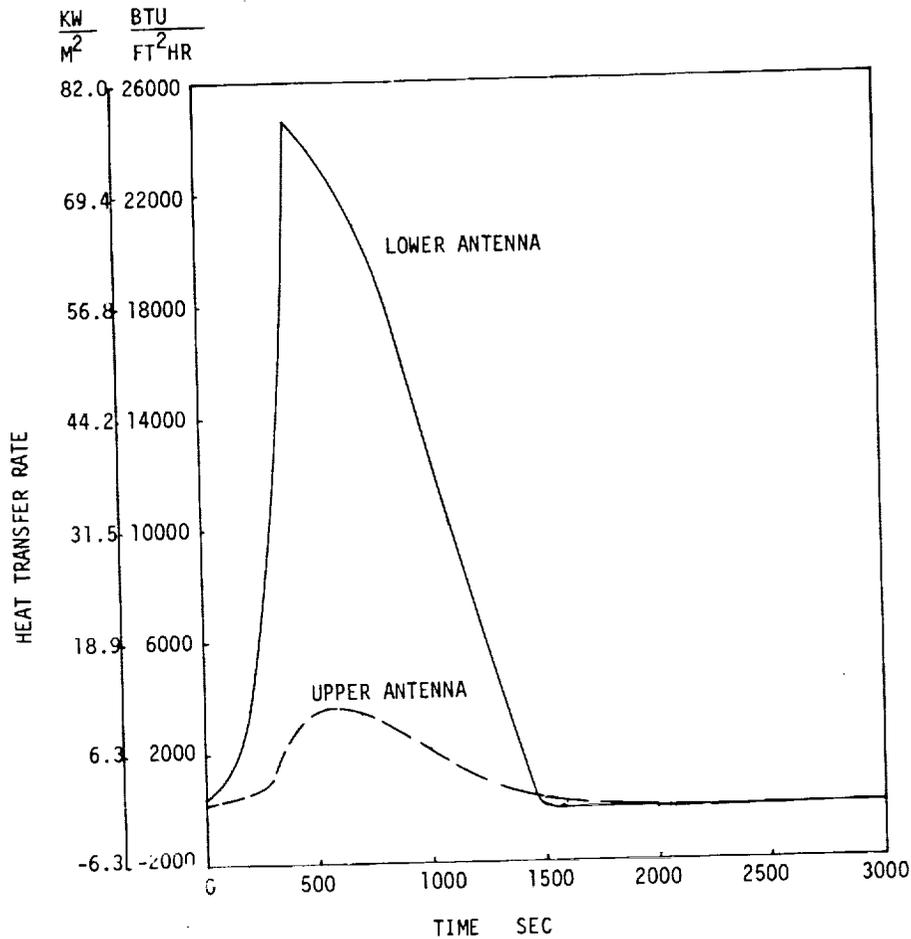


FIGURE 18 CALCULATED ANTENNA REGION TPS SURFACE CONVECTIVE HEAT TRANSFER RATES DURING ENTRY

The majority of the aerodynamic heat convection is reradiated by the TPS surface to space and to the atmosphere during entry. Prior to 1700 sec the antenna radiated to the space environment and the assumed sink temperature was 1.7 K (3°R). After this time when the vehicle is in the lower atmosphere a sink temperature of 300 K (540°R) was used.

Thermal Response

The boundary conditions and heat inputs discussed in the previous section were used as inputs to determine the transient thermal response of the antenna in orbit and during entry.

Orbital temperatures. - The response of the antenna to variations in the external surface heat flux is slow in orbital flight, due to the large mass

of the antenna, the TPS thickness and the isothermal spacecraft interior boundary condition. Therefore, large temperature gradients in the antenna during orbital flight are small.

The maximum and minimum orbital temperatures of the antenna components and their surroundings were calculated using the ANTENA thermal model with the extreme heating conditions shown on figure 12. The antenna was soaked at these conditions until the equilibrium temperatures were attained. The equilibrium temperatures for various antenna and TPS components are tabulated in table IV for the hot and cold extremes at both upper and lower antenna locations. Although the TPS surface temperature over the lower antenna varied between  $-66$  and  $128^{\circ}\text{C}$  ( $-86$  and  $262^{\circ}\text{F}$ ) the calculated extremes within the antenna are only  $-24$  and  $82^{\circ}\text{C}$  ( $-12$  and  $180^{\circ}\text{F}$ ). The corresponding TPS bondline temperatures are  $-25$  and  $82^{\circ}\text{C}$  ( $-13$  and  $180^{\circ}\text{F}$ ). The limits of the antenna temperature are approximately equal to the TPS bondline limits. The TPS surface temperature range over the upper antenna is  $-89$  to  $53^{\circ}\text{C}$  ( $-102$  to  $128^{\circ}\text{F}$ ) while the antenna temperature range is  $-57$  to  $47^{\circ}\text{C}$  ( $-70$  to  $117^{\circ}\text{F}$ ). The upper antenna is more responsive to the absorbed heat than the lower because of the thinner TPS. However, as was explained in this section under Thermal Model, the fraction of the irradiation absorbed is smaller for the upper TPS due to its lower solar absorptivity.

Figures 19 thru 22 show the orbital temperature histories of the TPS and internal insulation at the upper and lower antenna locations in the extreme hot and cold environments. From an initial temperature of  $38^{\circ}\text{C}$  ( $100^{\circ}\text{F}$ ) about eight orbits are required to reach equilibrium at the lower antenna in the hot environment and about fourteen in the cold environment. By comparison, about five orbits are required to reach equilibrium at the upper antenna for both hot and cold conditions. The cabin wall temperature was assumed to be  $33^{\circ}\text{C}$  ( $92^{\circ}\text{F}$ ) in all four of these analyses. Except for the outer portion of the TPS, very little temperature change occurs during an individual orbit in any component of the antenna. Therefore, the antenna temperature variations are not illustrated. However, the antenna temperatures calculated in these analyses were close to or between the values in figures 19 thru 22 labeled TPS bondline and TG-15000.

To assess the impact of the interior isothermal wall boundary condition on the antenna temperature, two cases were analyzed with hot and cold cabin walls. The first case consisted of the upper antenna in the cold orbit with a  $17^{\circ}\text{C}$  ( $62^{\circ}\text{F}$ ) wall and the second of the lower antenna in the hot orbit with a  $50^{\circ}\text{C}$  ( $122^{\circ}\text{F}$ ) wall. Table V summarizes these results. At the upper antenna, lowering the cabin temperature changed the antenna temperature from  $-56$  to  $-63^{\circ}\text{C}$  ( $-70$  to  $-82^{\circ}\text{F}$ ). At the lower antenna, the increase in cabin temperature resulted in an increase of the antenna equilibrium temperature of  $8^{\circ}\text{C}$  ( $14^{\circ}\text{F}$ ) to  $89^{\circ}\text{C}$  ( $193^{\circ}\text{F}$ ). Comparison of the equilibrium temperatures shown on tables IV and V shows that the interior boundary condition significantly affects the antenna temperature.

Another parameter which can change the antenna temperature is the power dissipated in the radiating elements by operation. To evaluate this effect an input power of fifty watts and an 85% element efficiency were assumed. Thus, 7.5 watts were dissipated in the radiating elements. To maximize the effect

TABLE IV  
ANTENNA AND TPS TEMPERATURES  
FOR HOT AND COLD ORBITAL ENVIRONMENTS

LOCATION	UPPER ANTENNA		LOWER ANTENNA	
	HOT	COLD	HOT	COLD
TPS SURFACE	128.	-102.	262.	-86.
RSI BONDLINE	124.	-96.	197.	-82.
TPS BONDLINE	118.	-70.	180.	-13.
TG-15000, OUTBOARD	116.	-47.	174.	-1.
<u>ANTENNA COMPONENTS</u>				
DUST COVER, SIDE	117.	-69.	179.	-12.
DUST COVER, BOTTOM	117.	-68.	178.	-11.
MOUNTING FLANGE	117.	-70.	179.	-12.
RADIATING ELEMENT WALL	117.	-70.	179.	-12.
RADIATING ELEMENT INSULATION	117.	-70.	179.	-12.
RADIATING ELEMENT RIDGES	117.	-70.	179.	-12.
CIRCUIT BOARD, OUTBOARD	117.	-70.	179.	-12.
CIRCUIT BOARD, INBOARD	117.	-70.	179.	-12.

1. TEMPERATURES IN °F
2. ANTENNA NOT TRANSMITTING
3. CABIN WALL TEMPERATURE, 92°F
4. VEHICLE ORIENTATION PER TABLE I.

of this heating, and since most of the power is fed to the center element, it was assumed that all of this energy was concentrated in the center element, evenly divided among the four ridges. Table VI shows the antenna equilibrium temperatures for continuous antenna operation in the cold upper and hot lower environments. Antenna operation results in a temperature increase of about 11°C (20°F) in each case. However, even with this internal heating, no large temperature gradients occur.

Temperatures of the upper and lower antennas during orbital thermal conditioning were also calculated. For these cases, the heating rates shown on figures 13 and 14 were impressed on the TPS surface and the cabin wall was assumed to be 33°C (92°F). Figure 23 shows the orbital variation of the equilibrium temperature of the lower RSI bondline and the antenna center element for the 559 W/m<sup>2</sup> (117 BTU/hr ft<sup>2</sup>) heating case. Although the outer TPS temperatures vary considerably during the conditioning orbit, the antenna temperature level is essentially constant. The larger temperature variation at the upper antenna location is due to the thinner RSI. Table VII shows the corresponding maximum and minimum temperatures of the antenna components and TPS during orbital thermal conditioning. The lower antenna temperature is constant at 48°C (119°F) while the upper antenna temperature varies less than ±1°C (±2°F) from 3°C (37°F).

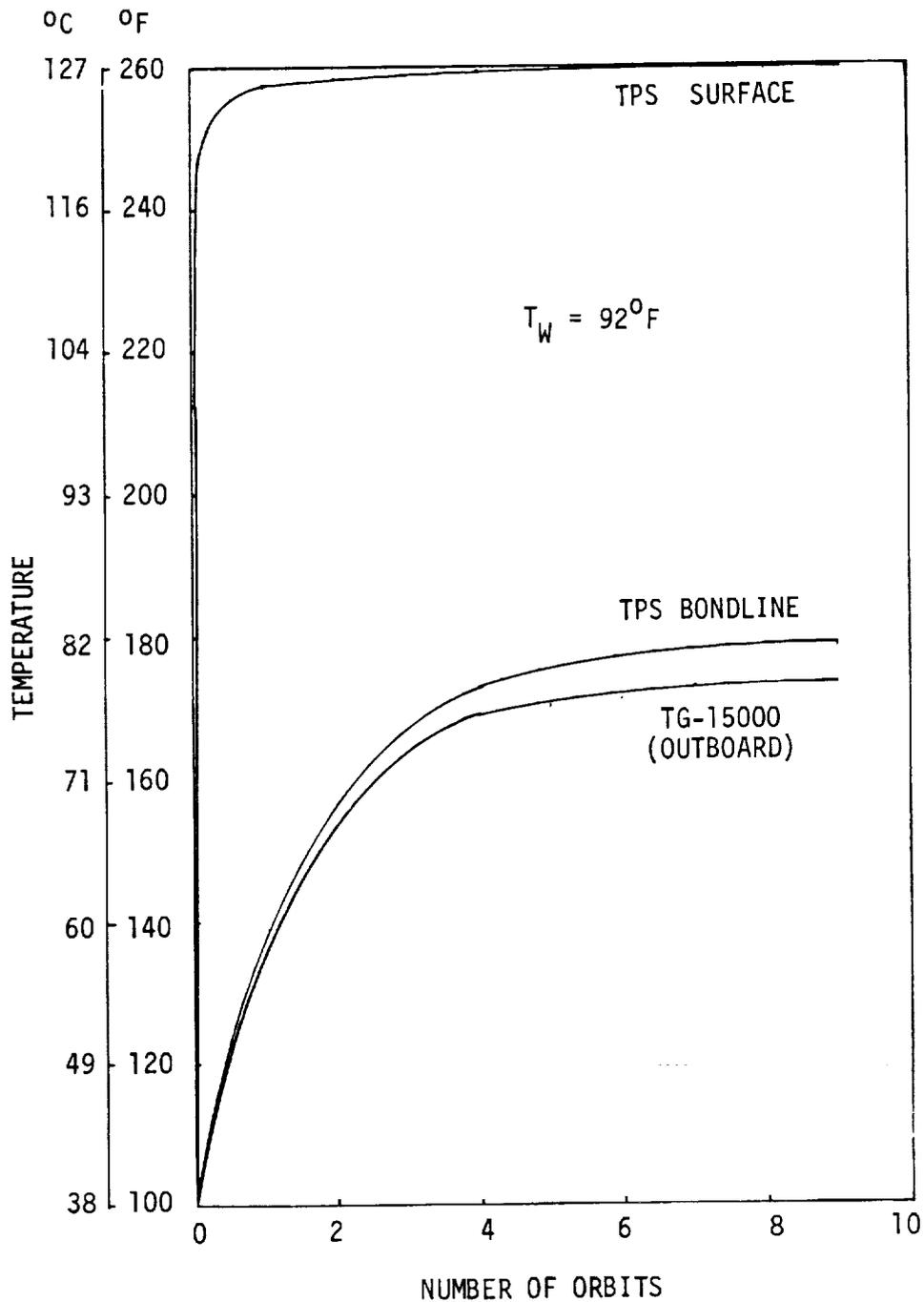


FIGURE 19 LOWER ANTENNA THERMAL RESPONSE - HOT ORBIT

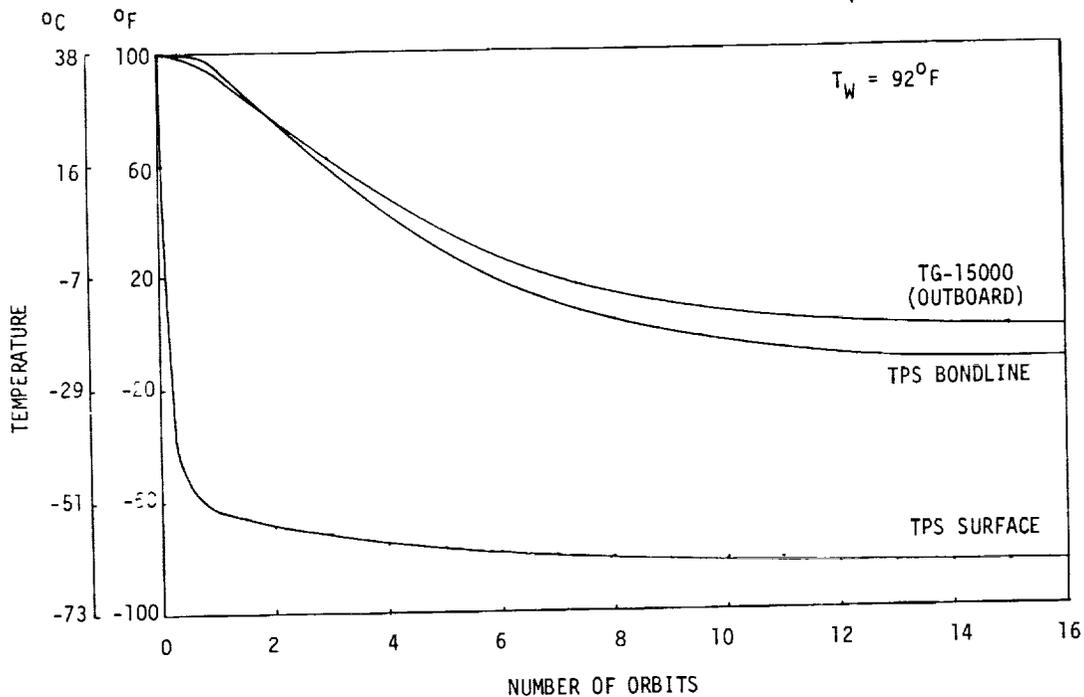


FIGURE 20 LOWER ANTENNA THERMAL RESPONSE - COLD ORBIT

Entry temperatures. - The TPS pressure and heat flux histories shown in figures 17 and 18 were used to determine the entry thermal response of the antenna. Four sets of initial conditions were used: the hot and cold orbital equilibrium temperatures for the upper and lower antennas as shown on table IV. The cabin wall temperature used for these calculations was 33°C (92°F).

Although entry from these extreme temperature environments is not contemplated, these analyses yield the maximum antenna temperature range. Entry from any other environment, such as that following the thermal conditioning orbit, will result in temperatures between these limits.

Table VIII presents the maximum temperature of each of the upper and lower antenna components following entry from the hot environment. These temperatures should be the maximum temperatures of the antennas for this trajectory. The maximum TPS bondline temperatures of the upper and lower antennas are 124 and 144°C (256 and 292°F), respectively. The maximum temperatures reached within the antenna are 100 and 128°C (212 and 262°F), respectively. The TPS bondline temperatures are less than the 177°C (350°F) maximum allowable due to structural heat leaks as was explained previously.

Figures 24 and 25 are the TPS temperature histories over the lower antenna during entry from the hot and cold environments. The initial temperature has little effect on the TPS surface temperature. However, the TPS bondline temperatures differ by about 69°C (125°F) after 4000 seconds. Due to heat soak effects, the maximum temperature occurs later in the trajectory for components farther from the surface. Therefore, the temperatures shown on

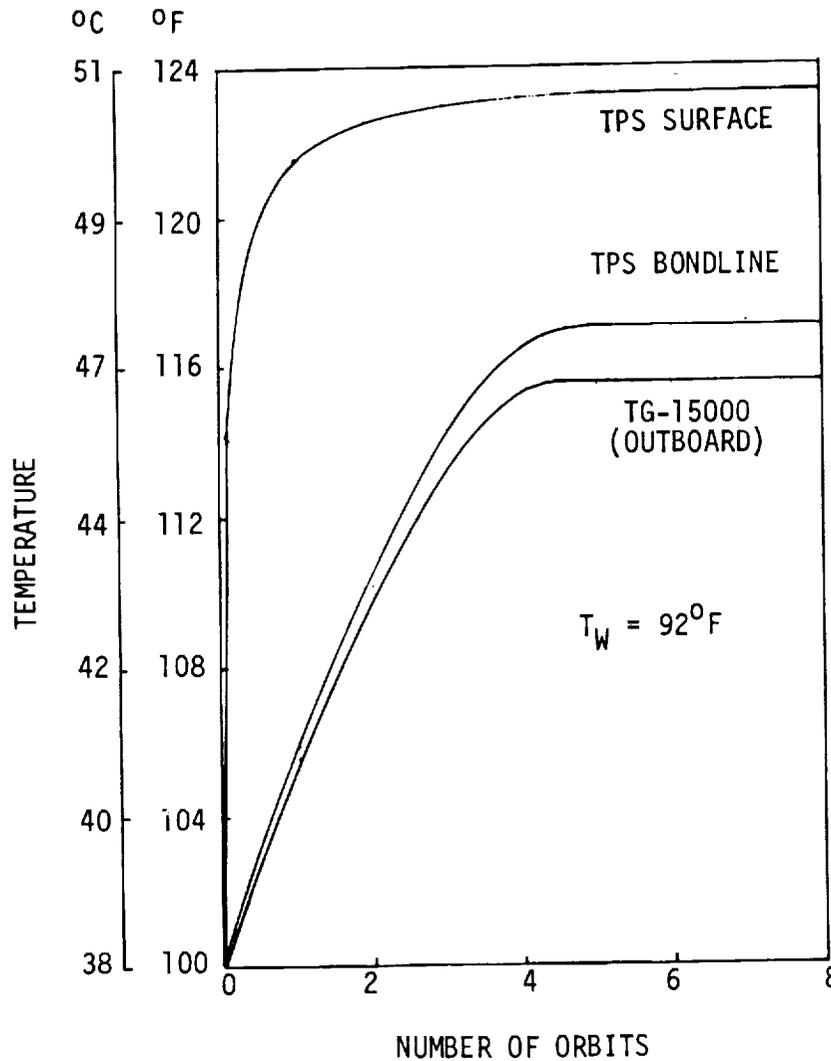


FIGURE 21 UPPER ANTENNA THERMAL RESPONSE - HOT ORBIT

table VIII are not concurrent. Also, due to the heat sink effect of the antenna the temperatures of the skin and TPS are not uniform laterally as shown by figure 26. Temperatures are shown at three locations on the RSI bondline for the initial cold and hot temperatures of the lower antenna. These locations are directly above the antenna and 28 and 58 cm (11 and 23 in.) from the antenna center (below nodes 324, 319 and 314 shown in figure 5). These locations were also selected for thermocouple, (T/C), installation in the thermal test fixture discussed later in this report. Due to the heat capacity of the antenna, the TPS temperature increases with distance from it. This variation is further illustrated on figure 27 in which isotherms on the RSI bondline surface are shown at the time of maximum RSI bondline temperature. The temperature increases with distance from the antenna but the rate of increase decreases with distance such that no effect of the antenna is found at large distances. The distortion of the isotherms from a circular shape is due to the

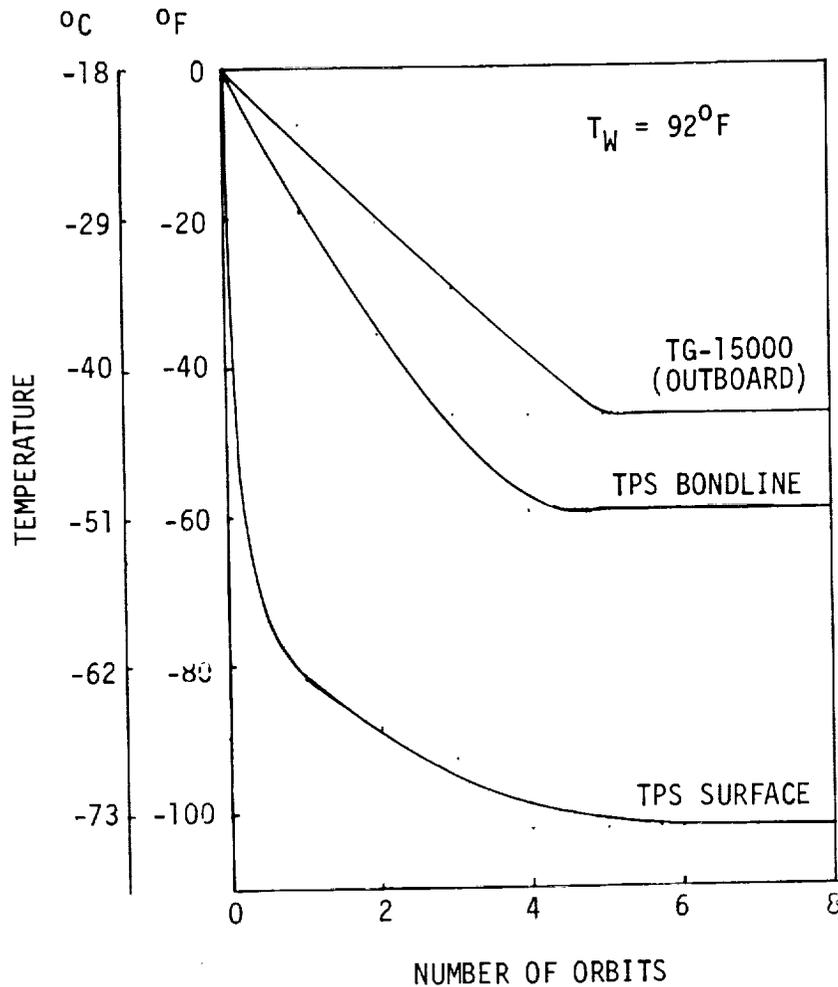


FIGURE 22 UPPER ANTENNA THERMAL RESPONSE - COLD ORBIT

preferential conduction path in the fore-and-aft direction as a result of the skin support stringers. Therefore the temperature gradient is smaller in the forward direction than it is laterally.

Figures 28 and 29 are the TPS temperature histories over the upper antenna during entry from the hot and cold environments. These temperatures are lower than those over the lower antenna but the response time of the interior of the TPS is shorter because of the thinner RSI. Also the effect of initial temperature is smaller because of the thinner RSI. The difference in the TPS bondline temperatures at 4000 sec is only about 33°C (60°F). Figures 30 and 31 show the time variation in temperature of three RSI bondline locations during entry while figure 27 shows the isotherms at the time of maximum temperature. The behavior of this surface is similar to that for the lower antenna bondline.

Figure 32 shows the temperature response inside the upper and lower antennas due to the entry heat flux for the different hot and cold initial conditions. These temperatures correspond to a location in the center of the

TABLE V

ANTENNA AND TPS TEMPERATURES WITH HOT AND COLD CABIN WALL

LOCATION	UPPER ANTENNA CABIN WALL = 62°F COLD ORBIT	LOWER ANTENNA CABIN WALL = 122°F HOT ORBIT
TPS SURFACE	-112.	263.
RSI BONDLINE	-95.	201.
TPS BONDLINE	-82.	193.
TG-15000, OUTBOARD	-62.	189.
<u>ANTENNA COMPONENTS</u>		
DUST COVER, SIDE	-81.	193.
DUST COVER, BOTTOM	-80.	192.
MOUNTING FLANGE	-82.	193.
RADIATING ELEMENT WALL	-82.	193.
RADIATING ELEMENT INSULATION	-82.	193.
RADIATING ELEMENT RIDGES	-82.	193.
CIRCUIT BOARD, OUTBOARD	-82.	193.
CIRCUIT BOARD, INBOARD	-82.	193.

1. TEMPERATURE IN °F
2. ANTENNA NOT TRANSMITTING
3. VEHICLE ORIENTATION PER TABLE I

TABLE VI

ANTENNA AND TPS TEMPERATURES WITH ANTENNA OPERATING

LOCATION	UPPER ANTENNA COLD ORBIT	LOWER ANTENNA HOT ORBIT
TPS SURFACE	-102.	263.
RSI BONDLINE	-76.	202.
TPS BONDLINE	-56.	198.
TG-15000, OUTBOARD	-37.	190.
<u>ANTENNA COMPONENTS</u>		
DUST COVER, SIDE	-54.	200.
DUST COVER, BOTTOM	-53.	196.
MOUNTING FLANGE	-54.	198.
RADIATING ELEMENT WALL	-51.	200.
RADIATING ELEMENT DIELECTRIC	-50.	200.
RADIATING ELEMENT RIDGES	-50.	200.
CIRCUIT BOARD, OUTBOARD	-53.	199.
CIRCUIT BOARD, INBOARD	-53.	198.

1. TEMPERATURES IN °F
2. CABIN WALL TEMPERATURE, 92°F
3. VEHICLE ORIENTATION PER TABLE I
4. POWER DISSIPATION = 1.875 WATT/RIDGE IN CENTER ELEMENT

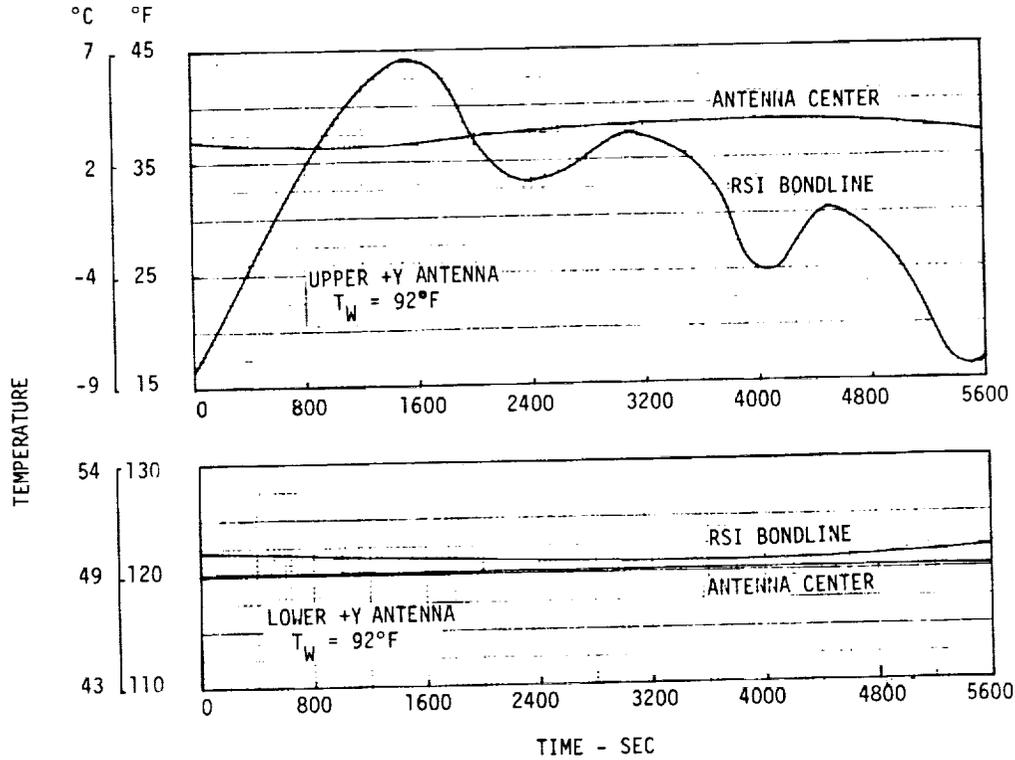


FIGURE 23 ANTENNA TEMPERATURES DURING ORBITAL THERMAL CONDITIONING

TABLE VII  
ANTENNA AND TPS TEMPERATURES  
DURING ORBITAL THERMAL CONDITIONING

LOCATION	UPPER ANTENNA		LOWER ANTENNA	
	MAX	MIN	MAX	MIN
TPS SURFACE	78.	-31.	198.	80.
RSI BONDLINE	45.	16.	123.	121.
TPS BONDLINE	38.	35.	119.	119.
TG-15000, OUTBOARD	43.	40.	118.	117.
<u>ANTENNA COMPONENTS</u>				
DUST COVER, SIDE	38.	36.	119.	118.
DUST COVER, BOTTOM	38.	36.	119.	119.
MOUNTING FLANGE	38.	36.	119.	119.
RADIATING ELEMENT WALL	38.	36.	119.	119.
RADIATING ELEMENT INSULATION	38.	36.	119.	119.
RADIATING ELEMENT RIDGES	38.	36.	119.	119.
CIRCUIT BOARD, OUTBOARD	38.	36.	119.	119.
CIRCUIT BOARD, INBOARD	38.	36.	119.	119.

1. TEMPERATURES IN °F
2. CABIN WALL TEMPERATURE, 92°F
3. LOWER +Y ANTENNA  
UPPER +Y ANTENNA
4. VEHICLE ATTITUDE PER TABLE I

TABLE VIII

ANTENNA AND TPS MAXIMUM ENTRY TEMPERATURES

LOCATION	UPPER ANTENNA	LOWER ANTENNA
TPS SURFACE	772.	1594.
RSI BONDLINE	389.	335.
TPS BONDLINE	256.	292.
TG-15000, OUTBOARD	225.	246.
<u>ANTENNA COMPONENTS</u>		
DUST COVER, SIDE	210.	261.
DUST COVER, BOTTOM	210.	257.
MOUNTING FLANGE	212.	262.
RADIATING ELEMENT WALL	206.	261.
RADIATING ELEMENT INSULATION	206.	261.
RADIATING ELEMENT RIDGES	206.	261.
CIRCUIT BOARD, OUTBOARD	206.	261.
CIRCUIT BOARD, INBOARD	203.	259.

1. TEMPERATURES IN °F
2. CABIN WALL TEMPERATURES, 92°F
3. ENTRY FROM MAXIMUM HOT CONDITIONS

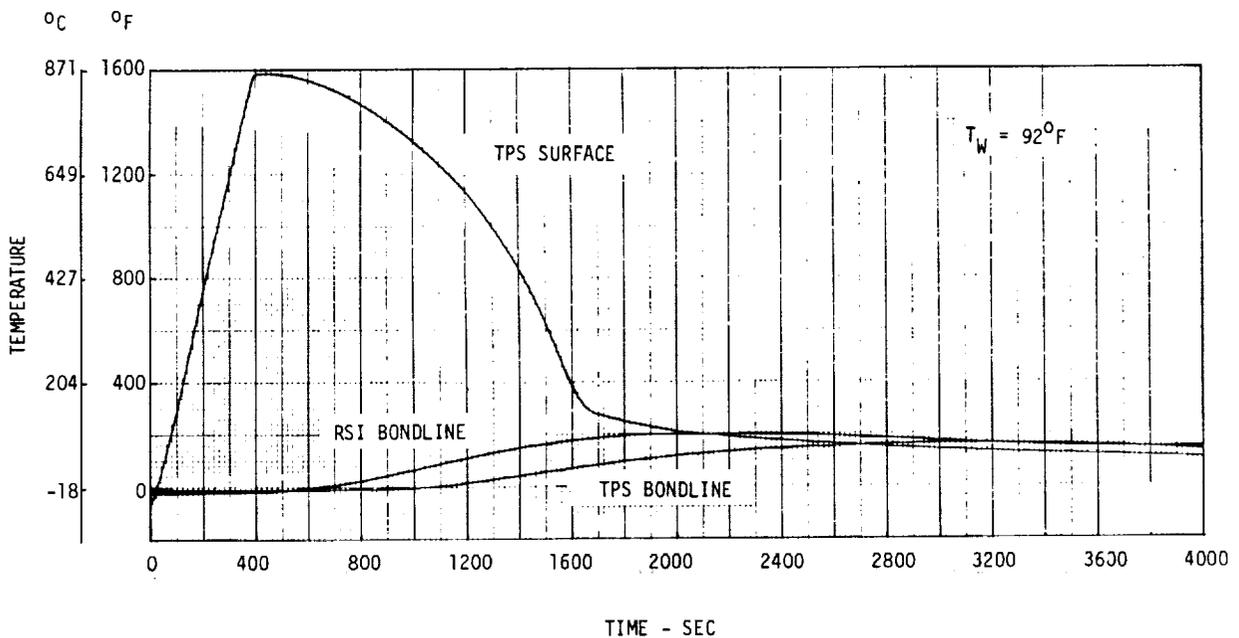


FIGURE 24 TPS TEMPERATURES DURING ENTRY AT LOWER ANTENNA - HOT ENVIRONMENT

**DEVELOPMENT OF S-BAND ANTENNA INTERFACE DESIGN**

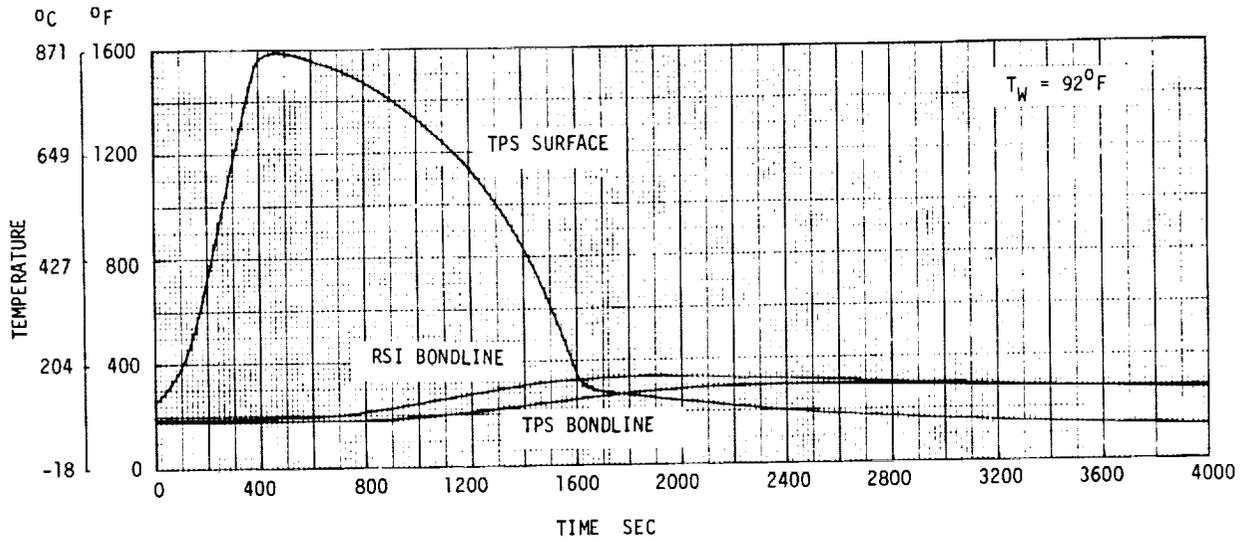


FIGURE 25 TPS TEMPERATURES DURING ENTRY AT LOWER ANTENNA - COLD ENVIRONMENT

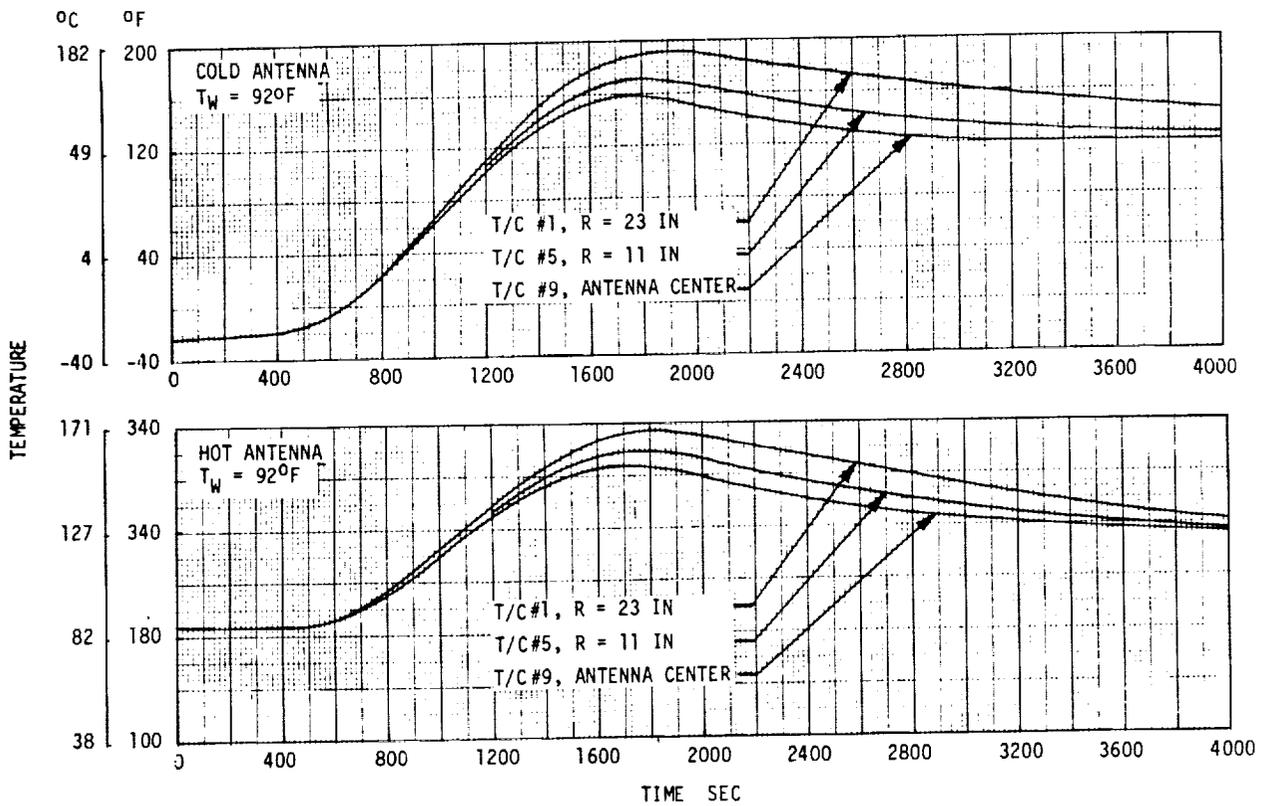


FIGURE 26 RSI BONDLINE TEMPERATURES DURING ENTRY AT LOWER ANTENNA

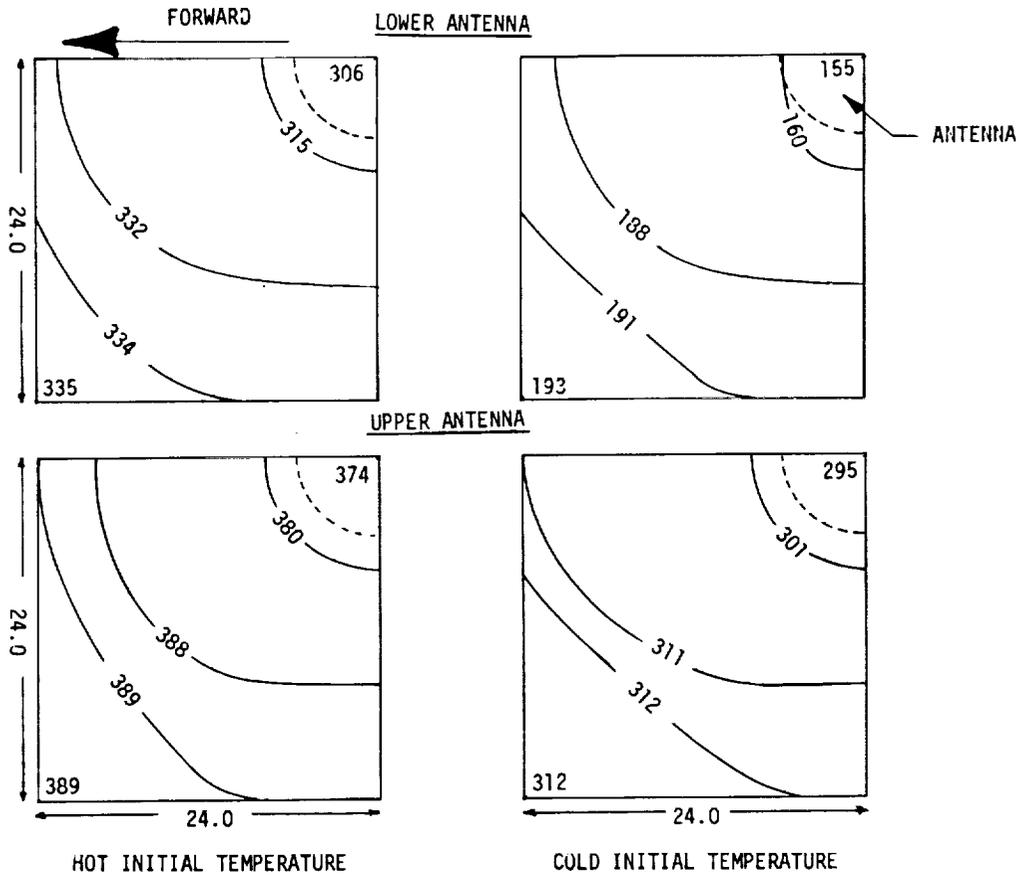


FIGURE 27 RSI BONDLINE ISOTHERMS AT MAXIMUM ENTRY TEMPERATURE

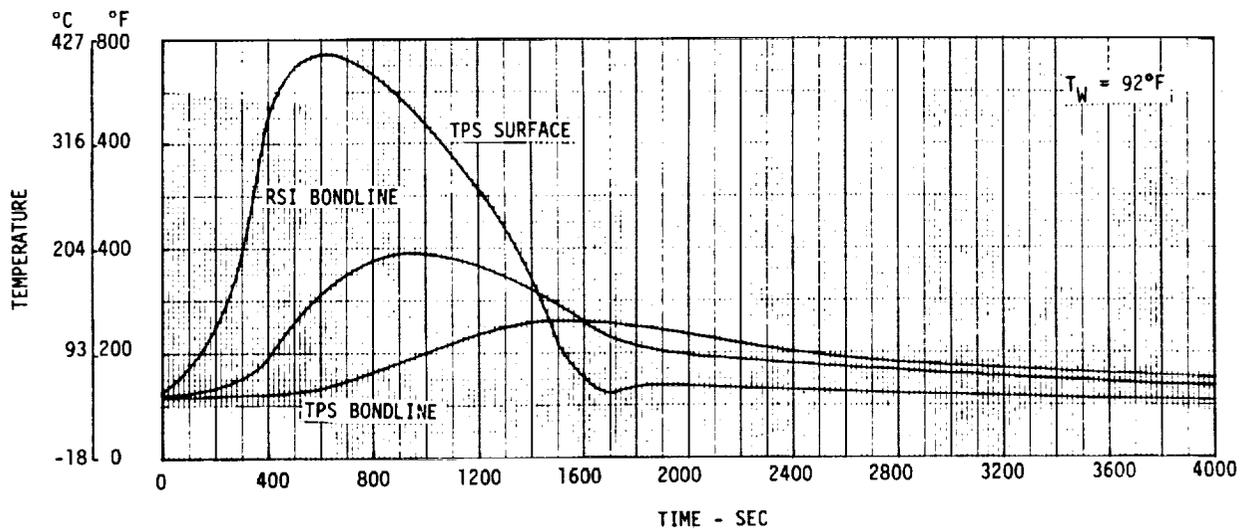


FIGURE 28 TPS TEMPERATURES DURING ENTRY AT UPPER ANTENNA - HOT ENVIRONMENT

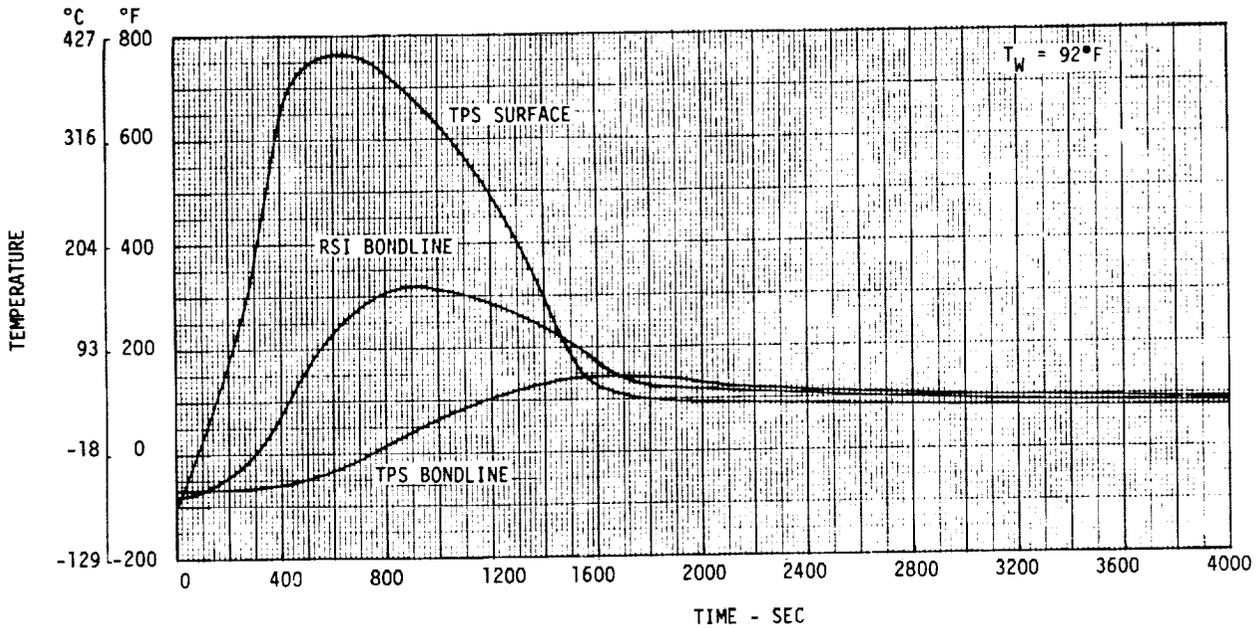


FIGURE 29 TPS TEMPERATURES DURING ENTRY AT UPPER ANTENNA - COLD ENVIRONMENT

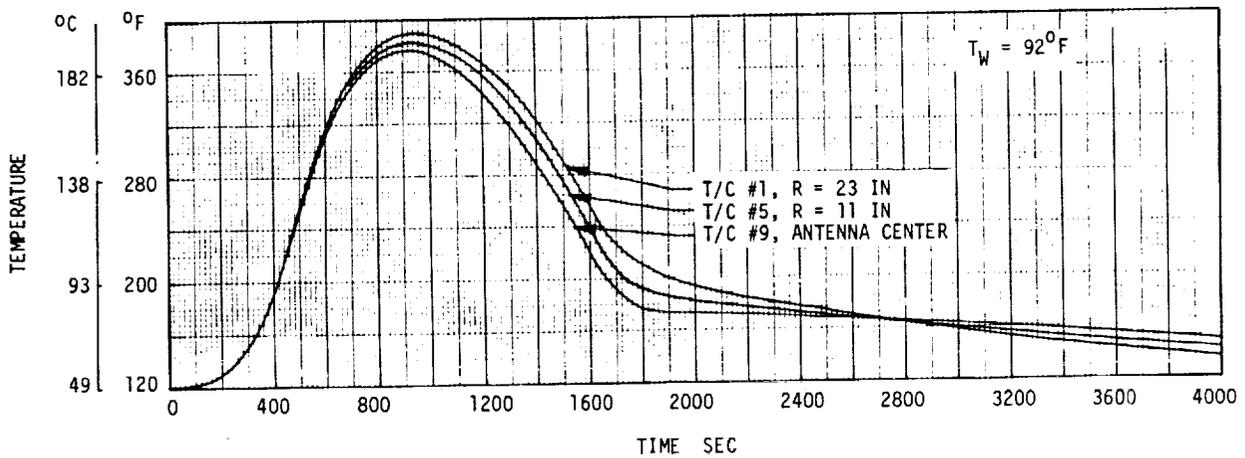


FIGURE 30 RSI BONDLINE TEMPERATURES DURING ENTRY AT UPPER ANTENNA - HOT ENVIRONMENT

antenna's center radiating element (node 32, figure 7). The upper antenna responds faster to heat flux variations but reaches a lower maximum temperature than the lower antenna.

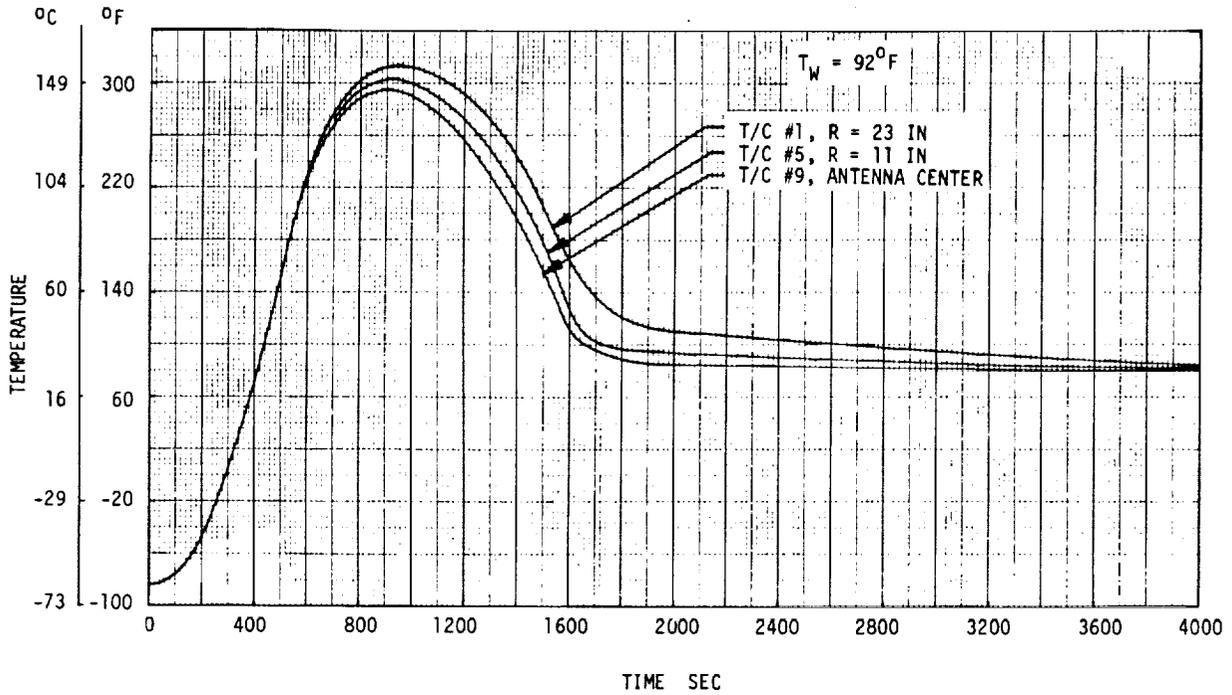


FIGURE 31 RSI BONDLINE TEMPERATURES DURING ENTRY AT UPPER ANTENNA - COLD ENVIRONMENT

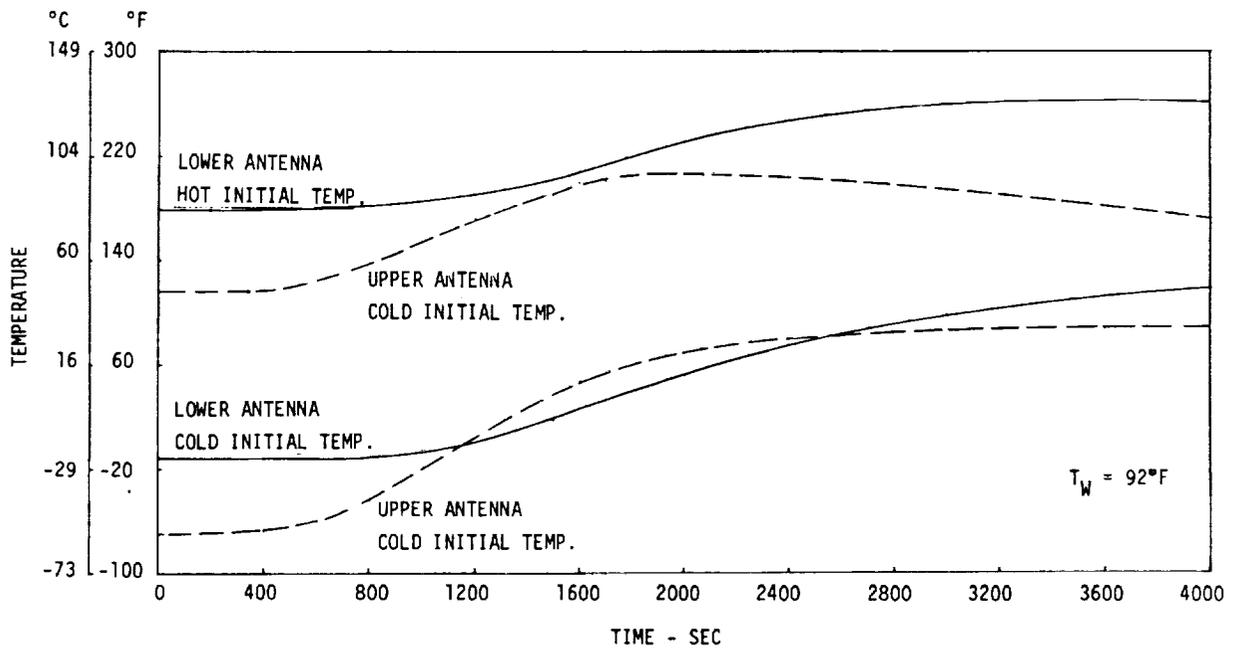


FIGURE 32 CENTER RADIATING ELEMENT TEMPERATURES DURING ENTRY

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**THERMAL TEST FIXTURE**

The Thermal Test Fixture task (Task 2.3.2) consisted of designing and fabricating a test fixture to be used to verify by thermal vacuum testing the thermal analyses and the thermal control plan obtained in Task 2.3.1 by evaluating the thermal response of the test fixture and the S-band antenna in simulated orbital and entry environments. The objectives of this task were: (1) to model a section of Space Shuttle Orbiter Structure with provisions for mounting the S-band antenna; (2) to minimize the effects of terminating the edges on the antenna's thermal response; (3) to provide a simulation of the interfaces with the space environment and the internal features of the Orbiter; and (4) to provide instrumentation for measuring the temperature histories at node points defined in the analytical thermal model. The details of the test fixture were reviewed with NASA prior to fabrication to ensure compatibility with NASA-JSC test facility requirements. This section discusses the design and fabrication approaches used to achieve these objectives.

**Design**

The thermal test fixture was designed to model the structure surrounding an S-band antenna so that both orbital and entry environments could be simulated and the thermal response of the antenna measured. The Orbiter features which could influence the antenna temperature and, thus, the test fixture design are: (1) the TPS over the antenna and surrounding skin; (2) the skin-stringer structure around the antenna; (3) the antenna mounting frame; (4) the internal insulation; (5) the crew cabin wall inboard of the internal insulation; and (6) the location of the antenna on the spacecraft. Other considerations which influence the test fixture design are: (1) test facility (vacuum chamber) size; (2) facility heating and cooling capabilities; and (3) instrumentation requirements.

One of the most important features considered was the exterior interface with the space and entry environments - the TPS. The TPS thicknesses at the upper and lower antenna locations (figure 1) are substantially different due to the difference in thermal environments as was discussed in the previous section, THERMAL ANALYSES - Thermal Model. This difference would require the construction of two thermal test fixtures or the necessity of having the capability of replacing the TPS in order to simulate the temperatures within both the upper and lower antennas. Also, as was shown in figures 24 and 28, the surface temperatures of the TPS are very high during entry. These temperature levels would require very large heating rates during testing. Therefore, since the primary purpose of the test fixture is to obtain the thermal response of the S-band antenna and since the thermal behavior of the RSI has been repeatedly verified by other tests, it was decided to eliminate the RSI from the test fixture and to control the SIP surface to simulate calculated orbital and entry temperature histories. Because of this design feature the same test fixture can be used to simulate either an upper or a lower antenna installation. In addition to eliminating the need for two test

fixtures, the heating rates and temperature limits required to simulate entry can be obtained with strip heaters at significantly lower input powers since the SIP temperature is much lower than the RSI surface temperature as shown in figures 24 and 28. The most significant difference between the thermal responses of the upper and lower antenna installations is the rate of temperature change. The upper antenna responds more rapidly because of the thinner RSI. However, the maximum and minimum temperatures are essentially the same. The temperature history of the RSI-SIP interface is one of the node temperatures calculated during the analysis of each environmental condition.

Inclusion of the SIP over the test fixture results in a significant simplification which should be noted. Thermal analyses show that the RSI bondline temperature is more uniform than the skin (TPS bondline) temperature due to the nonisotropic stringer conductance, (e.g., figure 27). However, since the SIP is essentially a thermal insulation material its surface temperature is difficult to measure and control accurately. This deficiency was overcome by including a thin sheet of stainless steel bonded on top of the SIP surface. This thin metal sheet provides a uniform lateral conductive path for the impressed heat and will allow for more accurate temperature control because of the fast response of the metal to heat input. The rapid control capability and the insulation provided by the SIP should result in less overshoot from the desired temperature history at the surface of the antenna and in the surrounding skin. The stainless steel skin over the SIP also provides a surface for attaching strip heaters which can be individually controlled in sectors to simulate solar or entry heating. To simulate space cooling the heaters can be turned off and the metallic surface permitted to radiate to the vacuum chamber shroud or to a low temperature sink directly over the test fixture surface.

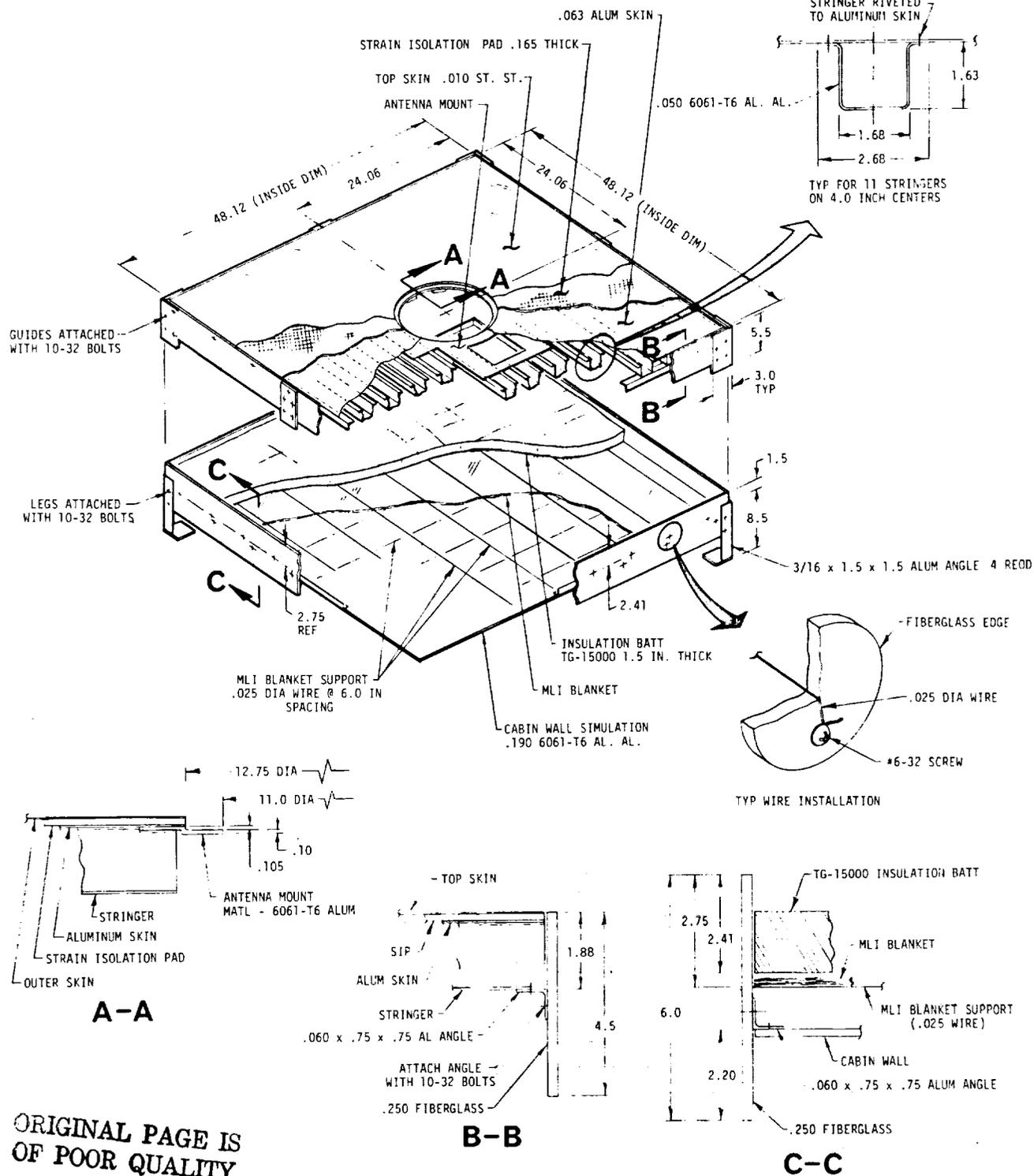
Another design feature considered was the test fixture size. The surface dimensions should be large enough to prevent the edge conditions from affecting the temperature distribution at the antenna. Thermal analysis showed that a 1.23 x 1.23 m (4.0 x 4.0 ft) panel was sufficient to provide approximately adiabatic edges. This size is also compatible with the NASA-JSC test facility vacuum chambers.

A sketch of the final thermal test fixture design is shown in figure 33. The design is based on the external surface simulation technique discussed above. The skin thickness is an average of five different skin thicknesses used in the antenna region on the Orbiter structure. The stringer cross-section and thickness are based on an average of several different stringer sizes in the immediate area of the Orbiter antenna. A coating is required on the internal side of the structure to provide the correct thermal radiation emittance characteristics.

Since the antenna mount is designed for installation of the antenna from outside of the Orbiter it was necessary to provide a removable circular panel of SIP and stainless steel that could be bonded to the antenna after installation in the test fixture.

# DEVELOPMENT OF S-BAND ANTENNA INTERFACE DESIGN

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30 APRIL 1976  
VOLUME I



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FIGURE 33 THERMAL TEST FIXTURE - DESIGN DETAILS

The simulated Orbiter structure, insulation and cabin wall panel are supported by a fiberglass frame. The frame was designed in two sections to permit easy access to the antenna following installation and to the internal insulation layers. The lower test fixture frame supports the TG-15000 insulation and MLI blanket over a panel which simulates the cabin wall.

During testing the test fixture must be surrounded on all four sides by at least 15 cm (6 in.) of min-K insulation or equivalent to provide adiabatic boundaries.

Instrumentation. - The thermal test fixture contains thirty-one 28-gage chromel-alumel thermocouples. An additional 17 thermocouples should be installed within the antenna. The thermocouples on the test fixture and those recommended on the antenna are tabulated in table IX. Figure 34 is a sketch of the thermocouple locations on the test fixture. A small hole is provided in the side of the test fixture frame for the thermocouple leads.

Temperature control. - It is recommended that thermocouples five through nine be designated as control points. Separation of the top surface into five zones (each quadrant and the center) will allow for more accurate temperature control of the test fixture surface. This zoning requires that heat be provided to each of the surface areas independently, by means of strip heaters, and that the temperature at thermocouples five through nine be used to regulate the heater power according to the predicted temperature history of these locations. Thus, locations one through four may vary slightly from the predicted temperature of the RSI bondline but the bondline temperature immediately over and around the antenna will have the correct input values.

#### Fabrication

The thermal test fixture was fabricated according to the design shown in figure 33, by the MDC Advanced Material Fabrication Facility. The test fixture is composed of two sections which can be separated for convenience. One section includes a skin-stringer structural panel and an antenna mounting frame. The other includes internal insulation and cabin wall simulations.

The basic frame (figure 35) was fabricated from 0.64 cm (0.25 in.) GSG fiberglass to minimize heat leaks between the upper and lower sections. The lower section includes an aluminum sheet to simulate the cabin wall. Two sets of orthogonal wires (figure 36) are used to support an MLI blanket ( $\sim$ 0.84 cm (0.33 in.) thick) and a layer of TG-15000 (3.81 cm (1.5 in.) thick) parallel to and 2.54 cm (1.00 in.) above the aluminum plate. The orthogonal wire configuration permits the insulation layer to be installed or removed without dismantling the test fixture.

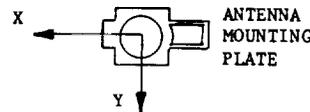
The structural section is composed of stringers and an antenna mounting fixture; both are riveted to a 1.23 x 1.23 m (4.00 x 4.00 ft) skin section. Figure 37 shows a bottom view of the skin-stringer panel. The stringer side of the panel was coated with a green epoxy primer (DeSoto 515-005 Super Koropon

TABLE IX  
ANTENNA THERMAL TEST FIXTURE INSTRUMENTATION LOCATIONS

NUMBER	COMPONENT	LOCATION	
1	Top skin (9)	X=16, Y=16 Top	
2		X=16, Y=-16	
3		X=-16, Y=-16	
4		X=-16, Y=16	
5		X=8, Y=8	
6		X=8, Y=-8	
7		X=-8, Y=-8	
8		X=-8, Y=8	
9		Center	
10	Al skin (16)	X=8, Y=-8 Top	
11		X=8, Y=8	
12		X=-8, Y=8	
13		X=-8, Y=-8	
14		X=0, Y=-16	
15		X=16, Y=0	
16		X=0, Y=16	
17		X=-16, Y=0	
18		X=24, Y=-24	
19		X=24, Y=24	
20		X=-24, Y=24	
21		X=-24, Y=-24	
22		X=16, Y=-16	
23		X=16, Y=16	
24		X=-16, Y=16	
25		X=-16, Y=-16	
26		TG-15000 (2)	X=12, Y=0 Top
27			TG-15000
28			X=-12, Y=0
29	Cabin Wall (4)	X=12, Y=12	
30		X=12, Y=-12	
31		X=-12, Y=-12	
32	F/G Antenna cover (1)	Center Top	
33	Antenna flange (2)	r=5.0, $\theta=22.5$ Top	
34	Antenna flange	r=5.0, $\theta=202.5$	
35	Dust Cover flange (2)	z=-1.88, $\theta=22.5$ Inside	
36	Dust Cover flange	z=-1.88, $\theta=202.5$	
37	Dust cover bottom (1)	Center Outside	
38	Circuit Board (1)	Center Bottom	
39	Center element wall(2)	z=-2.0, $\theta=45$	
40	Center element wall	z=-2.0, $\theta=225$	
41	+X element wall (2)	z=-2.0, $\theta=45$	
42	+X element wall	z=-2.0, $\theta=225$	
43	+Y element wall (2)	z=-2.0, $\theta=45$	
44	+Y element wall	z=-2.0, $\theta=225$	
45	-X element wall (2)	z=-2.0, $\theta=45$	
46	-X element wall	z=-2.0, $\theta=225$	
47	-Y element wall (2)	z=-2.0, $\theta=45$	
48	-Y element wall	z=-2.0, $\theta=225$	

1. Reference axes located at antenna center (top of element)

- +X forward
- +Z outboard (top)
- +Y forms right-hand system



2. Angles ( $\theta$ ) defined as positive rotation +X + +Y

3. All thermocouples chromel-alumel 28 gage with 10 foot leads  
(from test fixture exit)

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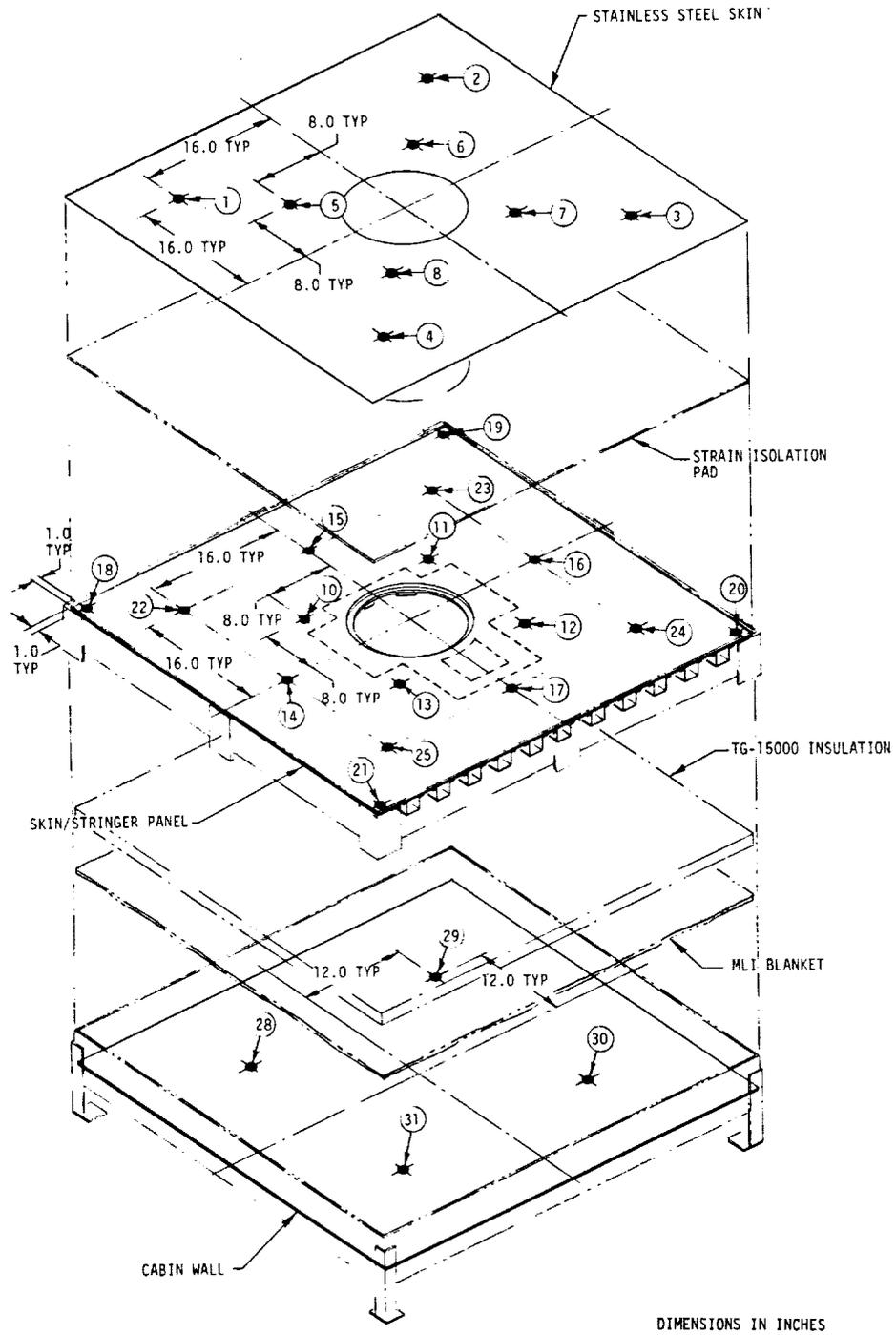
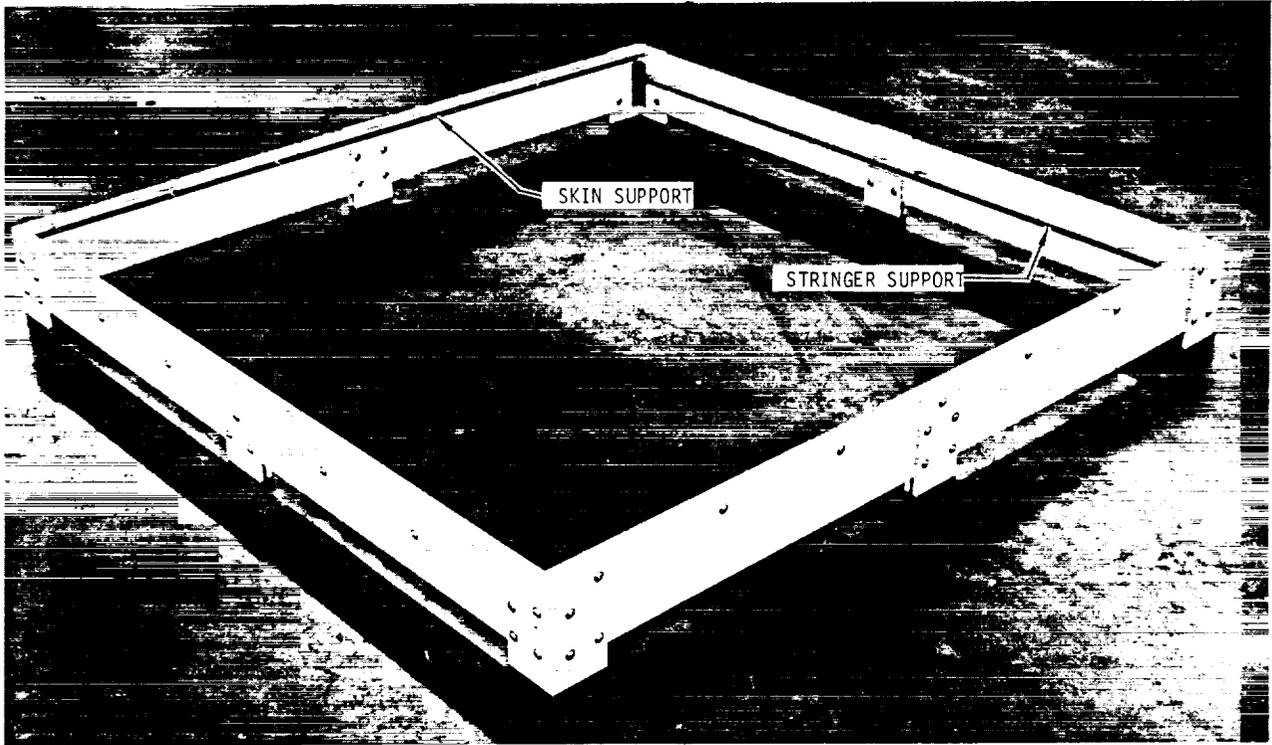
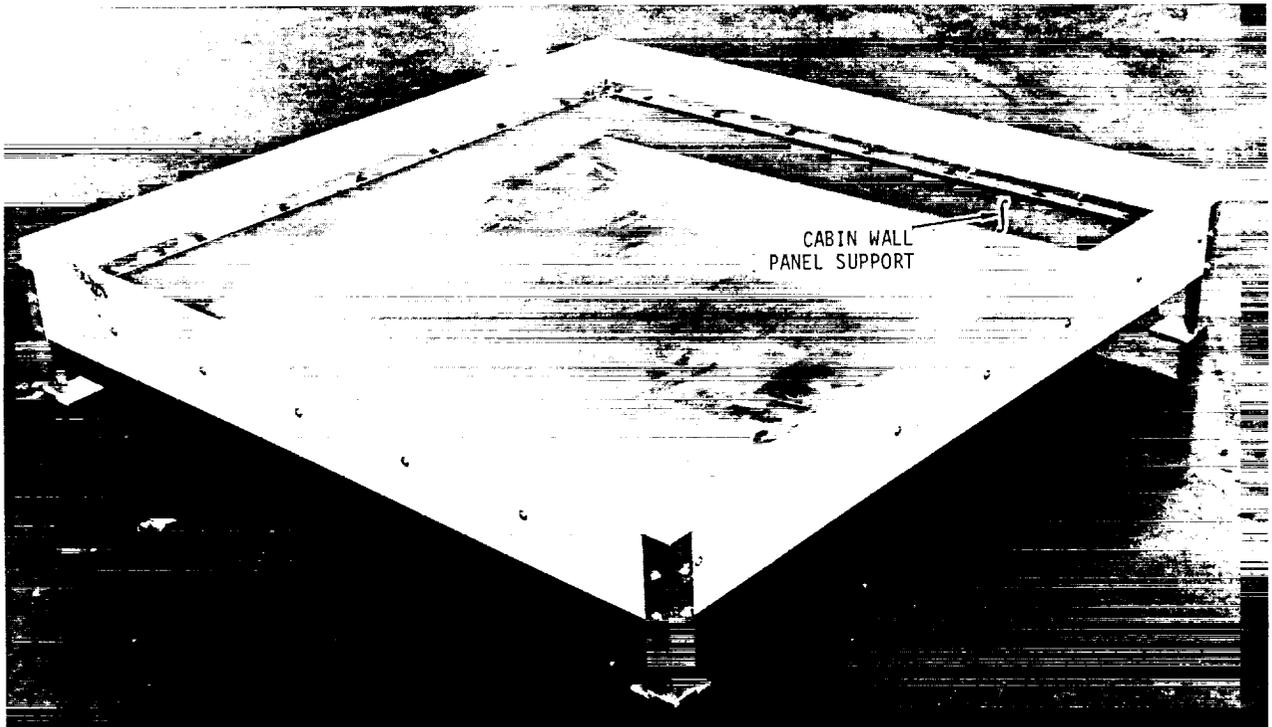


FIGURE 34 THERMOCOUPLE LOCATIONS



(a) UPPER SECTION



(b) LOWER SECTION

FIGURE 35 THERMAL TEST FIXTURE FRAME

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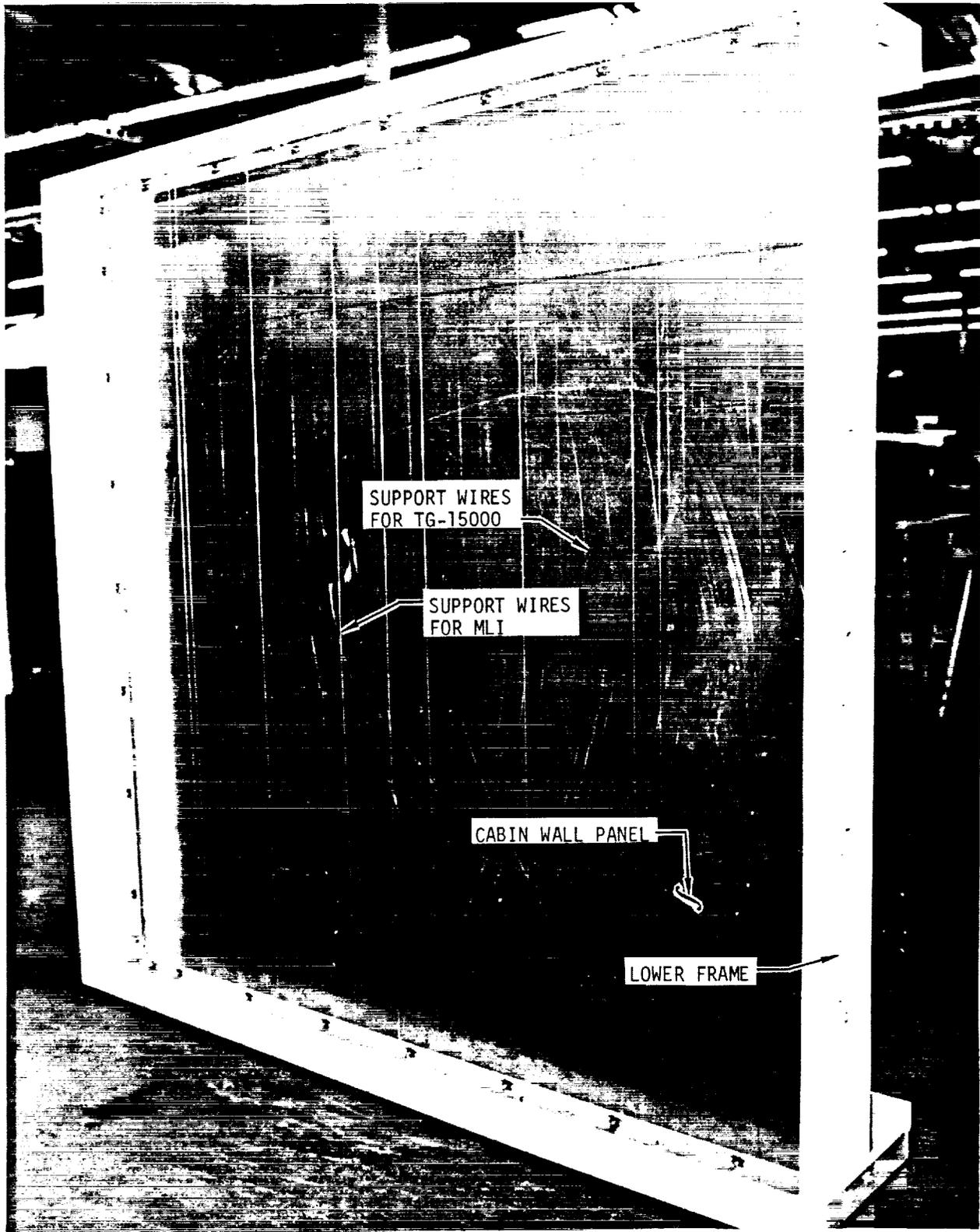


FIGURE 36 INTERNAL INSULATION SUPPORT

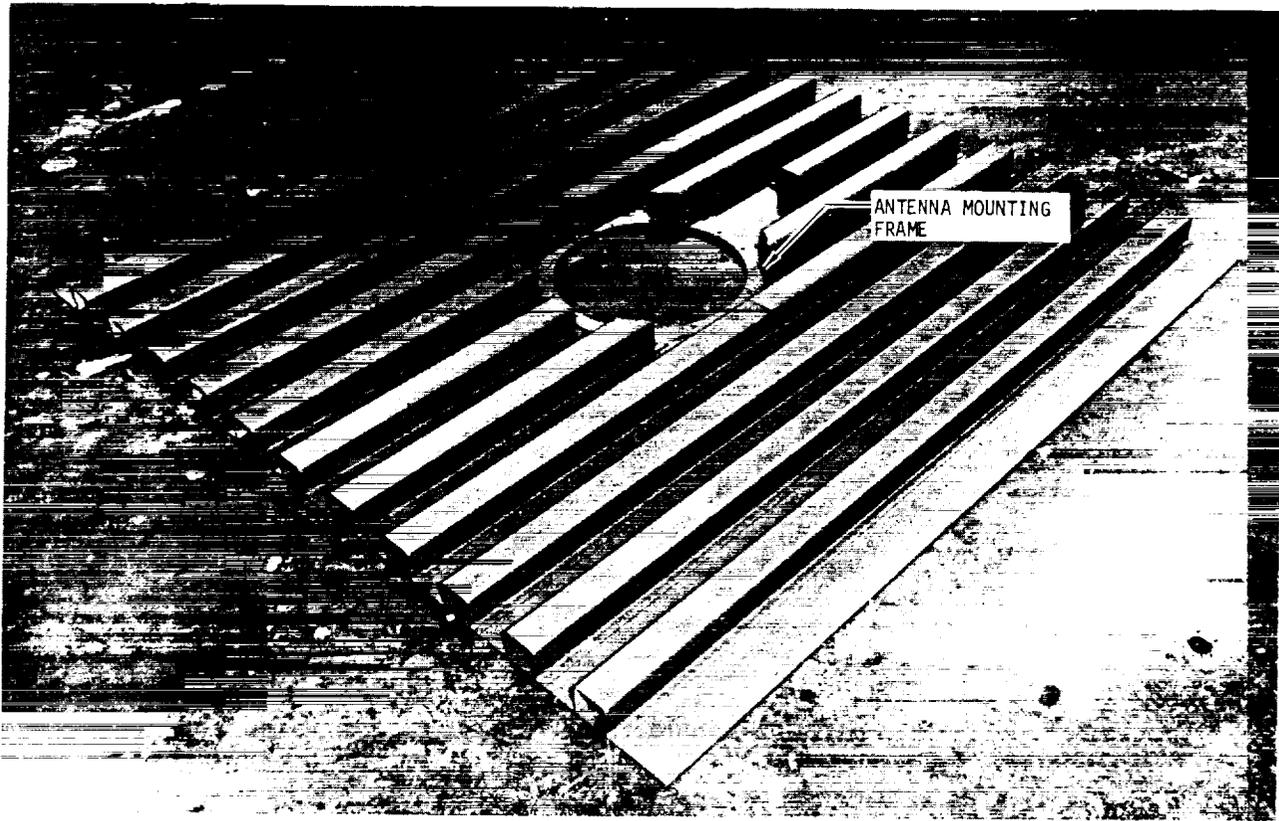


FIGURE 37 SKIN-STRINGER PANEL - BOTTOM VIEW

Fluid Resistant Primer) to provide the proper thermal emittance. This panel was attached to the angle brackets as shown in figure 35a. Figure 38 shows a top view of the skin-stringer panel attached to the test fixture framework. A layer of SIP (strain isolator pad) was bonded to the skin with RTV-560. A 0.254 mm (0.010) stainless steel sheet was bonded on top of the SIP with RTV-560 as shown in figure 39. A section was cut-out over the antenna through the SIP and stainless steel to permit installation of the antenna from the top. This circular section must be bonded to the antenna surface after installation to provide proper heat conduction during testing. The thin stainless steel surface serves as a conducting surface for heating the test fixture. Strip heaters can be bonded to this surface or it can be heated with quartz lamps and controlled to the RSI/SIP interface temperature.

The primary control thermocouples for the thermal test fixture are shown in figure 39. These thermocouples permit independent control of four areas surrounding the antenna and one directly over the antenna. Thermocouple instrumentation for monitoring the skin-stringer structural temperatures are shown in figure 40. The thermocouple leads were extended through holes in the stringers and skin and bonded to the skin surface. Figure 41 shows the control thermocouple installation on the panel used to simulate the cabin wall.

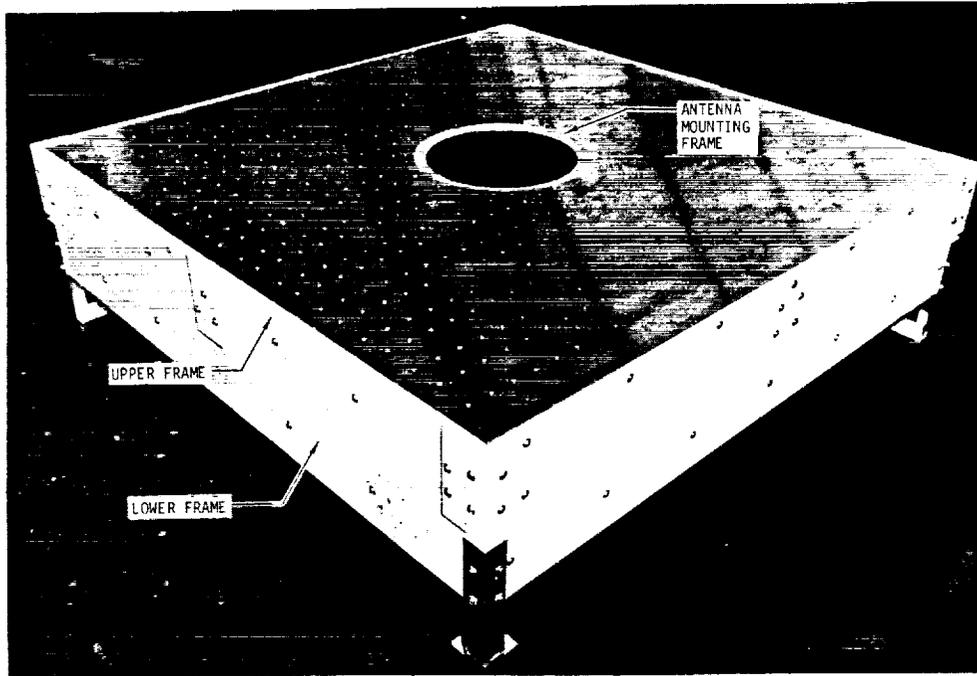


FIGURE 38 SKIN-STRINGER PANEL MOUNTED IN THERMAL TEST FIXTURE FRAME - TOP VIEW

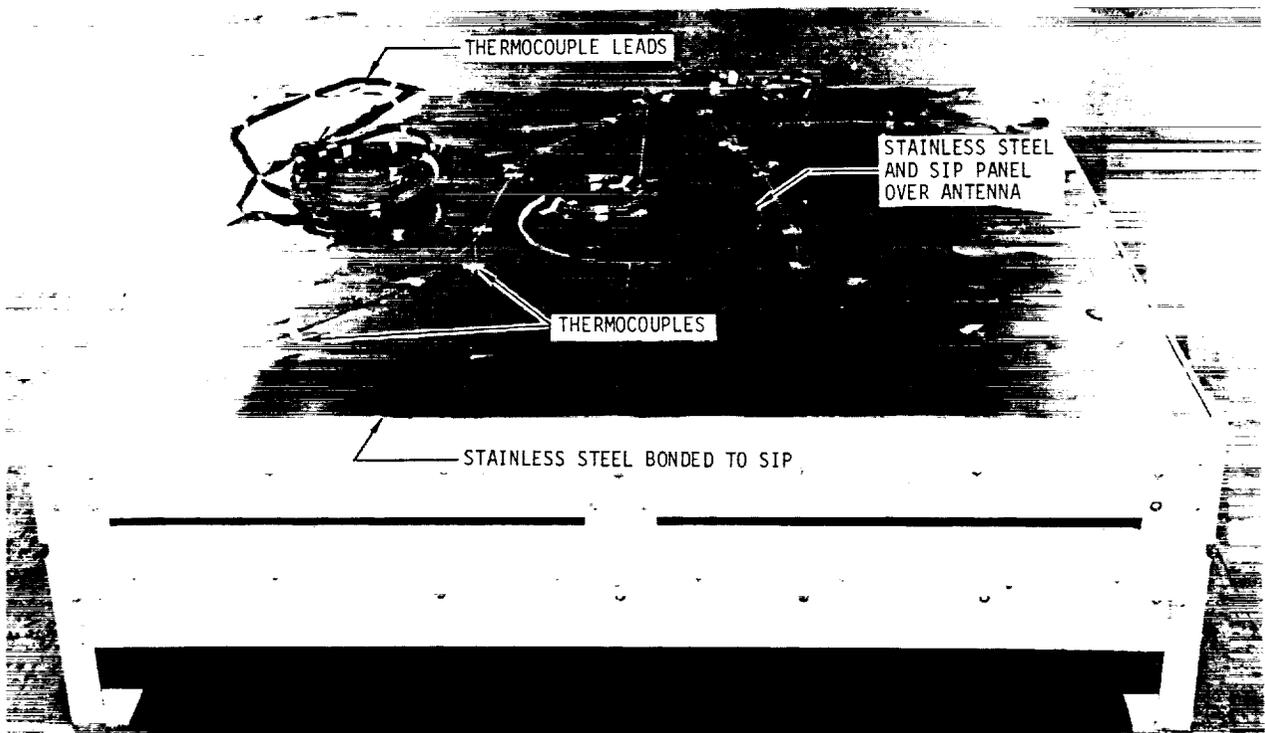


FIGURE 39 COMPLETED THERMAL TEST FIXTURE WITH SURFACE THERMOCOUPLES

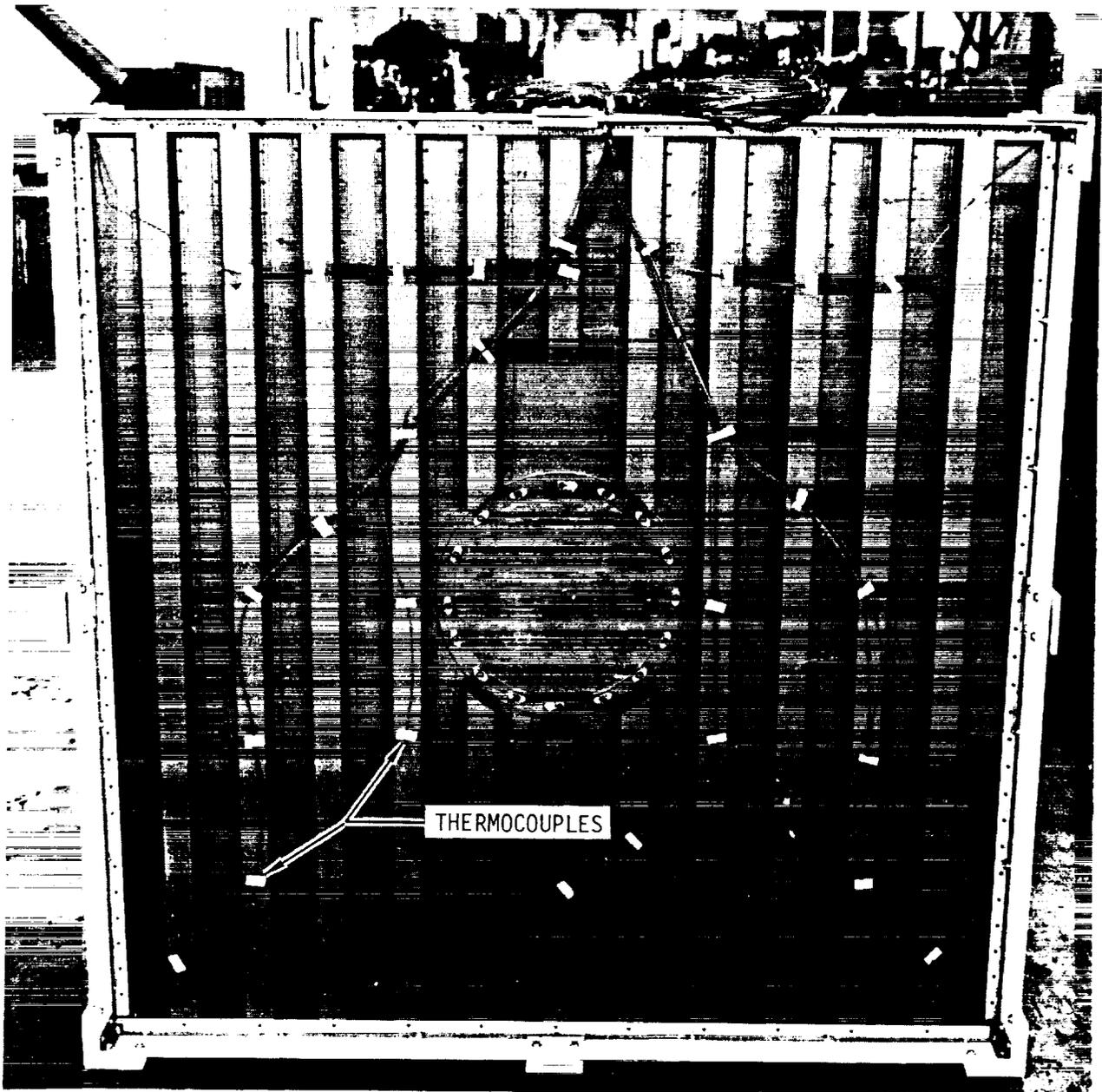


FIGURE 40 THERMOCOUPLE INSTALLATION ON SKIN-STRINGER PANEL

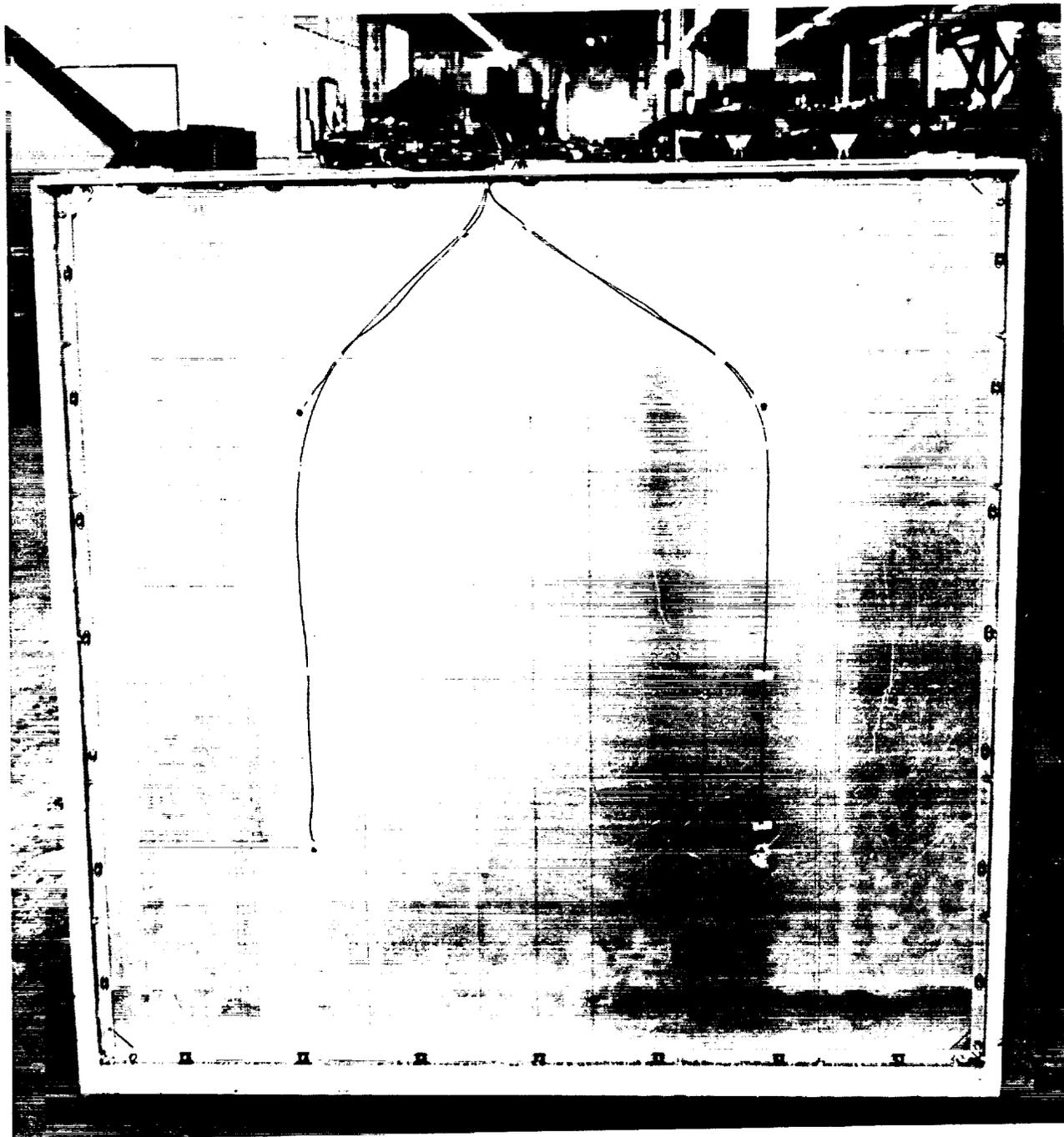


FIGURE 41 THERMOCOUPLE INSTALLATION ON CABIN WALL PANEL

**ELECTRICAL TEST FIXTURES**

The Electrical Test Fixtures task (Task 2.3.3) consisted of designing and fabricating two different electrical test fixtures to be used for evaluating the radiation pattern characteristics of an S-band antenna. The objectives of this task were to: (1) simulate sections of large and small radius spacecraft external surface (i.e., ground plane surface seen by the array antenna); (2) simulate the electrical characteristics of the thermal protection system (TPS) which is typically bonded to these surfaces; (3) select a size and/or edge configuration which minimizes the effects of the test fixture edges on the antenna radiation patterns; and (4) provide a mounting arbor configuration compatible with NASA-JSC antenna positioner requirements. This section discusses the design and fabrication approaches used to achieve these objectives.

**Design**

The electrical test fixtures were designed to simulate the surface areas surrounding two antenna locations on the Space Shuttle Orbiter. Simulation of the Orbiter TPS is the most significant problem associated with designing the test fixtures. In terms of antenna performance, the TPS on the Orbiter is similar to covering the antenna and the ground plane surrounding it with a dielectric sheet. The dielectric constant, loss tangent and thickness of the sheet cause transmission loss and support surface wave excitation which effect the radiation pattern and impedance of the antenna.

Figure 42 shows a sketch of the Orbiter and indicates the surface areas surrounding the two antenna locations used for this program along with the interface mold line (IML) (i.e., structural skin surface lines) shapes as a function of body station ( $X_0 = 505$  through  $X_0 = 600$ ). The antenna location on the upper body is at  $X_0 = 551.15$ ,  $Y_0 = +70.70$  and  $Z_0 = 472.50$  (Orbiter coordinates in inches) and is in an area that has a large transverse surface radius. The antenna location on the lower body is near the lower chine at  $X_0 = 556$ ,  $Y_0 = +95.62$  and  $Z_0 = 294.65$  and is in an area that has a small transverse surface radius. A right side configuration was arbitrarily selected for the two test fixtures. Since the Orbiter is symmetrical about the  $Y_0 = 0$  plane, the test results will be equally applicable to the left side with the proper coordinate interpretation.

The TPS thickness dimensions differ significantly around the upper and lower body antenna locations. The upper body TPS is generally less than 1.27 cm (0.5 in.) and the TPS on the lower body varies from 3.58 cm (1.41 in.) to 7.52 cm (2.96 in.). Only the RSI thickness is varied to obtain the required TPS thickness.

The test fixture design approach is shown in figure 43. Plywood frames 1.27 cm (0.5 in.) thick with a maximum spacing of 50.8 cm (20 in.) provide the surface contour control. Aluminum "T" stringers tie the frames together and

# DEVELOPMENT OF S-BAND ANTENNA INTERFACE DESIGN

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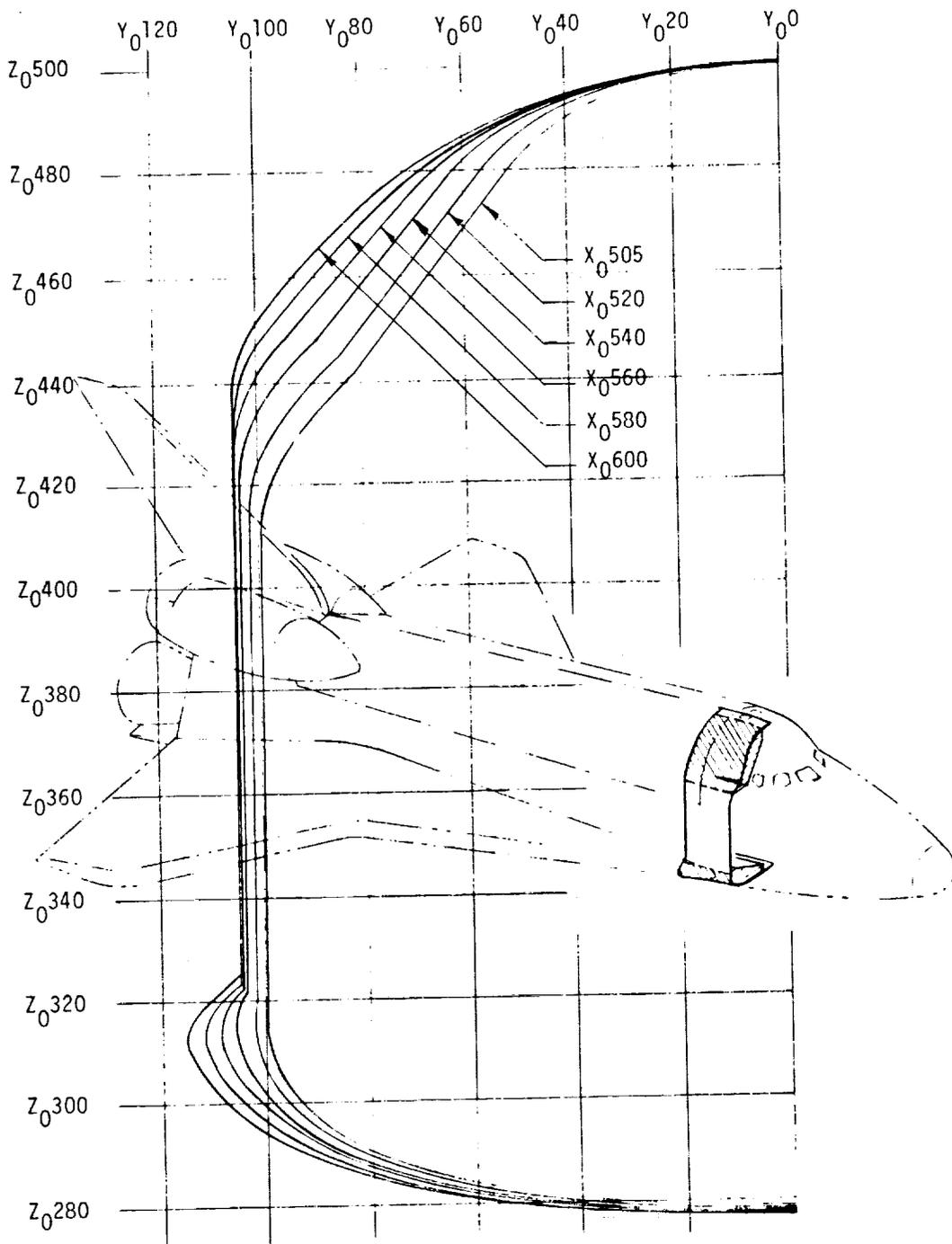


FIGURE 42 SPACE SHUTTLE ORBITER INTERFACE MOLDLINE CONTOURS

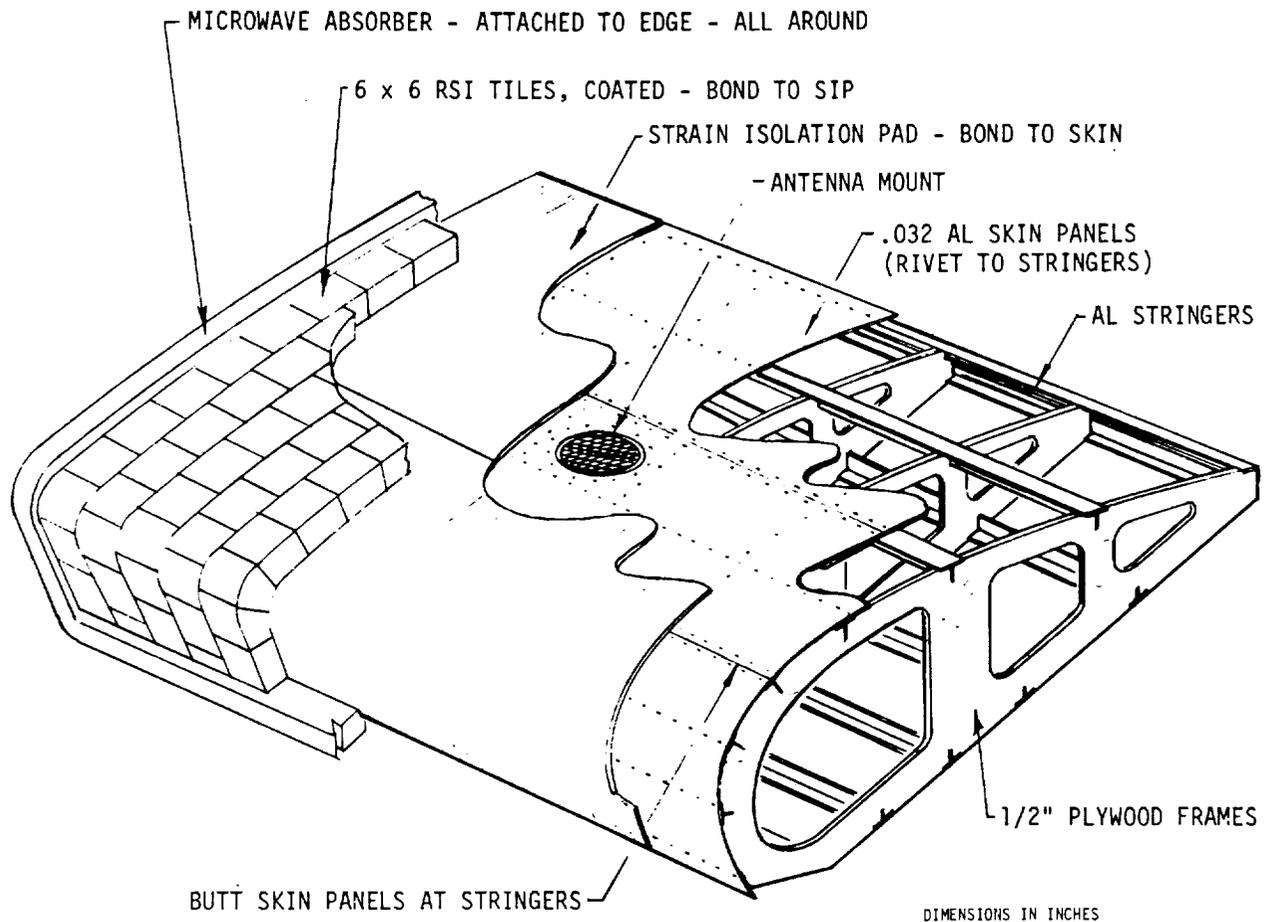


FIGURE 43 ELECTRICAL TEST FIXTURE - DESIGN DETAILS

provide intermediate support for an 0.813 mm (0.032 in.) aluminum skin. Simulated TPS is bonded to this skin to complete the simulation of the Orbiter surface configuration. Microwave absorber is attached to the test fixture edges to minimize the edge effects.

Size. - The size of the test fixtures was influenced by the cost and weight of the support structure required to maintain a rigid surface during antenna radiation pattern measurements and the requirement to minimize the edge effects on the test antenna radiation patterns. These factors resulted in the selection of a test fixture 2.44 m (8.0 ft) in length and a width based on the largest frame size which could be cut from a 1.22 x 2.44 m (4.0 x 8.0 ft) sheet of plywood. This approach also provides the largest test fixture that could be fabricated from standard material sizes within the budgeted cost. A larger test fixture would have increased both the weight and cost significantly due to the need for splicing and added bracing and the accompanying labor costs. Larger test fixtures would also have resulted in the need for longer transmission distances in order to obtain plane wave illumination.

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The size cited above results in test fixture edges approximately 9 wavelengths (at the lowest S-band antenna frequency -2217.5 MHz) from the center of the antenna.

Surface contours. - The Orbiter IML from body stations  $X_0 = 505$  to  $X_0 = 600$  (figure 42) bracket the center of the respective antenna locations at  $X_0 = 551.15$  and  $556$ . Based on the frame size limits discussed in the preceding subsection, the large and small radius test fixture IML's are bounded by  $Y_0 = 30$  and  $Z_0 = 440$ , and  $Y_0 = 30$  and  $Z_0 = 320$ , respectively.

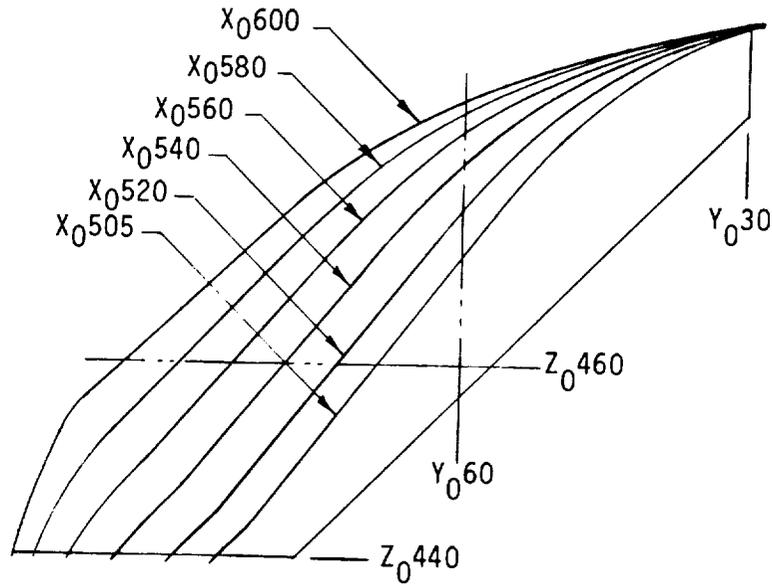
A simple cardboard model of each test fixture was constructed to aid in planning the frame configuration below the surface, the construction jig requirements and the overall test fixture details. These models showed that the use of the Orbiter body IML's would result in a compound surface curvature. However, a compound surface would require the use of form blocks to obtain a smooth skin. Since the use of form blocks would result in a significant construction cost increase, an alternate approach was used.

From the cardboard model, it was determined that a simply curved surface (i.e., one that can be covered with a flat sheet curved in only one direction) could be obtained that would not deviate from the IML's shown in figure 42 by more than 12.7 cm (5.0 in.) in one small region. In most locations the deviation would be less than 2.5 cm (1.0 in.). This is due to the fact the longitudinal radius of curvature is relatively large. By the use of the exact IML's for the frame contours adjacent to the antenna, the antenna aperture plane orientation can be held essentially coincident with that of an antenna normally installed in the Orbiter. The minor deviation from the exact Orbiter IML is expected to have no significant effect on the radiation patterns.

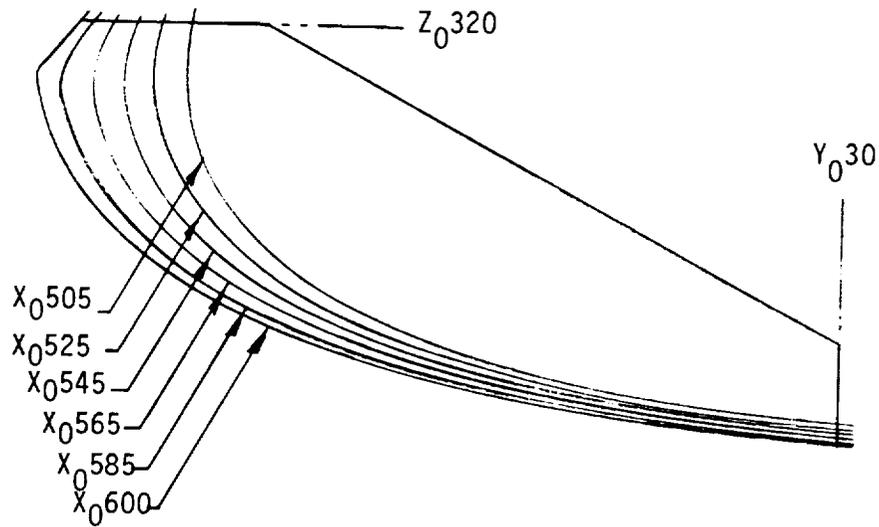
The test fixture frame contours were developed using a computer aided digital plotting machine. The Orbiter IML's adjacent to the antenna locations were used as control contours. Figure 44 shows the test fixture contours as developed. The variation in frame station selection results in antenna locations which are centered between the center frames of each test fixture. The maximum normal deviation from the true Orbiter IML is 2.7 cm (1.5 in.) and 1.3 cm (0.5 in.) for the large and small radius test fixtures, respectively. Full-scale frame lines were drawn individually on mylar for use in fabricating the respective frames. Figure 45 shows a trimetric view of each of the test fixture surfaces. These surfaces can be covered with flat skin sheets, thus, avoiding special tooling.

Interface definition: The coordinates of the test fixture forward and aft frames and the inboard and outboard edges are given in tables X thru XIII. These coordinates define the edge contours required on adjacent test fixtures in order to obtain a smooth transition between the test fixtures fabricated in this program. The coordinates are consistent with those shown in figures 44 and 45.

TPS simulation. - The TPS consists of two principal components and an adhesive as shown in figure 2. The RSI is fabricated in 15.2 x 15.2 cm (6.0 x 6.0 in.) tiles with the thickness dependent upon the location on the



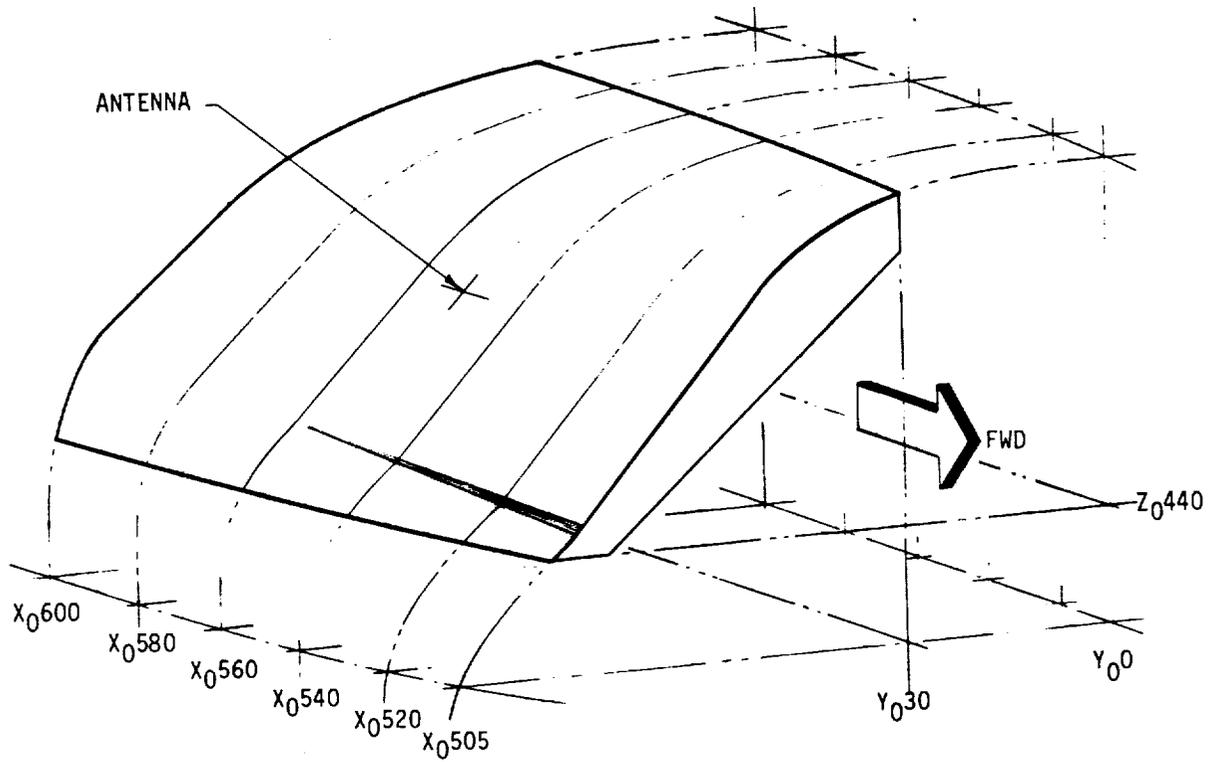
(a) LARGE RADIUS CURVATURE



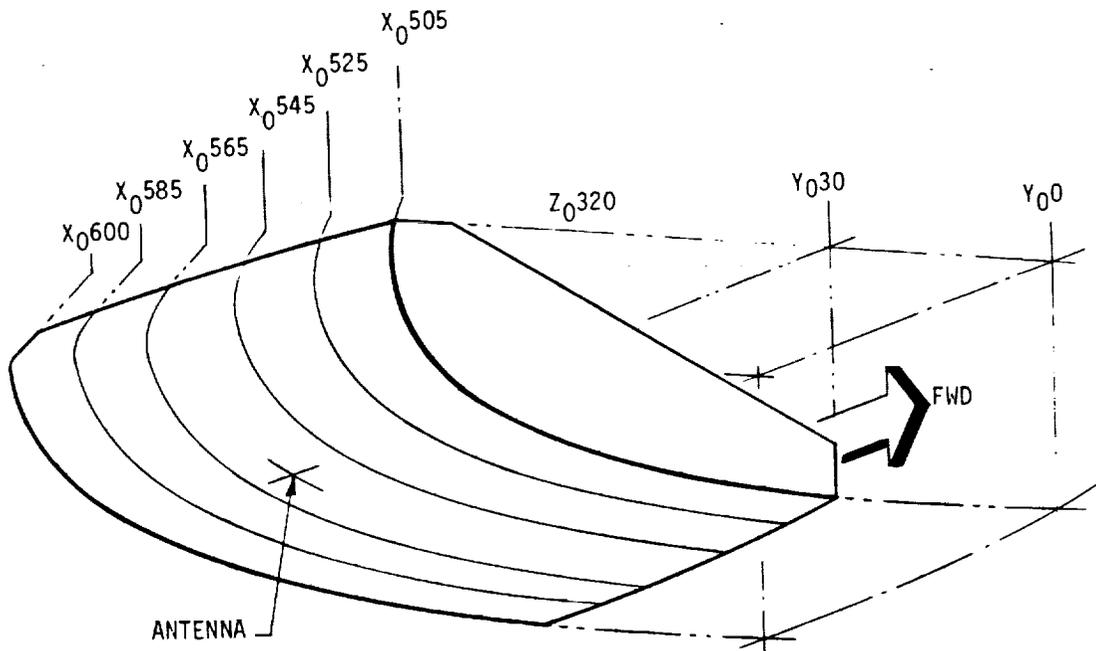
(b) SMALL RADIUS CURVATURE

FIGURE 44 ELECTRICAL TEST FIXTURE CONTOURS

DEVELOPMENT OF S-BAND ANTENNA INTERFACE DESIGN



(a) LARGE RADIUS CURVATURE



(b) SMALL RADIUS CURVATURE

FIGURE 45 ELECTRICAL TEST FIXTURE SURFACE CONTOURS - TRIMETRIC VIEW

TABLE X  
LARGE RADIUS TEST FIXTURE FRAME COORDINATES

Forward Frame $X_0 = 505$		Aft Frame $X_0 = 600$	
$Y_0$	$Z_0$	$Y_0$	$Z_0$
26.200	279.476	30.000	277.536
29.379	279.678	33.166	277.736
32.544	279.910	36.321	277.961
35.695	280.174	39.465	278.214
38.833	280.472	42.598	278.496
41.955	280.805	45.717	278.817
45.060	281.177	48.824	279.149
48.148	281.589	51.917	279.525
51.217	282.046	54.996	279.935
54.266	282.549	58.059	280.382
57.293	283.104	61.106	280.869
60.296	283.715	64.135	281.397
63.273	284.385	67.147	281.969
66.221	285.123	70.138	282.588
69.138	285.933	73.108	283.259
72.021	286.824	76.056	283.984
74.864	287.806	78.978	284.770
77.663	288.891	81.874	285.619
80.413	290.091	84.740	286.540
83.105	291.425	87.574	287.539
85.729	292.916	90.371	288.625
88.274	294.594	93.128	289.808
90.721	296.497	95.840	291.101
93.046	298.685	98.499	292.520
95.212	301.243	101.097	294.084
97.159	304.311	103.624	295.821
98.775	308.150	106.064	297.766
99.476	310.960	108.397	299.969
99.855	314.166	110.590	302.507
99.811	317.259	112.594	305.501
99.370	320.246	114.315	309.175
98.537	323.129	115.380	313.052
		114.407	316.028
		112.469	318.335
		110.480	320.606
		108.900	322.395

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TABLE XI  
LARGE RADIUS TEST FIXTURE  
INBOARD AND OUTBOARD COORDINATES

X <sub>0</sub>	Y <sub>0</sub> = 30	Z <sub>0</sub> = 440
	Z <sub>0</sub>	Y <sub>0</sub>
505	496.081	85.357
510	496.200	86.967
515	496.297	88.558
520	496.378	90.126
525	496.446	91.664
530	496.504	93.164
535	496.553	94.620
540	496.595	96.021
545	496.631	97.359
550	496.663	98.628
555	496.691	99.817
560	496.715	100.924
565	496.736	101.944
570	496.756	102.877
575	496.773	103.726
580	496.789	104.492
585	496.803	105.183
590	496.815	105.802
595	496.827	106.354
600	496.837	106.818

Orbiter. The tile pattern is offset 7.6 cm (3.0 in.) in one direction and aligned in the other. The gap width between tiles is nominally  $1.27 \pm .51$  mm ( $0.050 \pm 0.020$  in.).

The RSI was simulated with ECCOFOAM PS-A because of the high cost and limited availability of the actual material. The RSI (LI-900) dielectric constant and loss tangent are approximately 1.15 and .0008, respectively (reference 1). The ECCOFOAM is a light weight artificial dielectric plastic foam material, manufactured by Emerson & Cuming, Inc. It can be obtained with dielectric constants ranging from 1.1 to 2.0. The loss tangents of this material are approximately 0.0005 for dielectric constants up to 1.6. ECCOFOAM is relatively easy to cut into the required tile shape. The borosilicate TPS surface coating was simulated with a silicon dispersion coating (Dow Corning 92-009) which has a dielectric constant of 2.75 and a loss tangent of 0.0023. The borosilicate has a dielectric constant of about 2.45 and a loss tangent of about 0.0065 at the S-band frequencies (reference 2) according to test data from the Massachusetts Institute of Technology Laboratory for Insulation Research. Therefore, the Dow Corning silicon is considered a suitable simulation for the borosilicate coating material.

TABLE XII  
SMALL RADIUS TEST FIXTURE FRAME COORDINATES

Forward Frame $X_0 = 505$		Aft Frame $X_0 = 600$	
$Y_0$	$Z_0$	$Y_0$	$Z_0$
28.855	496.453	29.004	497.022
31.696	495.484	31.979	496.456
34.482	494.370	34.938	495.846
37.208	493.098	37.882	495.188
39.867	491.646	40.808	494.478
42.331	490.049	43.716	493.713
44.781	488.243	46.603	492.889
47.129	486.338	49.470	492.001
49.339	484.301	52.313	491.043
51.414	482.134	55.130	490.009
53.391	479.873	57.920	488.893
55.307	477.554	60.680	487.687
57.187	475.200	63.407	486.384
59.046	472.826	66.098	484.973
59.809	471.841	68.749	483.443
	Str Line	71.356	481.782
81.806	443.392	72.850	480.751
84.051	442.267	75.249	478.931
86.252	439.115	77.510	476.959
88.402	436.928	79.712	474.920
		81.896	472.863
		84.074	470.798
		86.248	468.730
		88.421	466.661
		88.800	466.300
			Str Line
		100.100	455.704
		101.861	453.190
		103.243	450.517
		104.458	447.773
		105.531	444.970
		106.424	442.091
		106.832	439.700

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TABLE XIII  
SMALL RADIUS TEST FIXTURE  
INBOARD AND OUTBOARD EDGE COORDINATES

$X_0$	$Y_0 = 30$	$Z_0 = 320$
	$Z_0$	$Y_0$
505	279.721	99.422
510	279.605	100.138
515	279.489	100.850
520	279.374	101.554
525	279.257	102.249
530	279.141	102.936
535	279.026	103.615
540	278.911	104.284
545	278.795	104.942
550	278.680	105.588
555	278.565	106.219
560	278.451	106.835
565	278.336	107.435
570	278.222	108.016
575	278.107	108.577
580	277.992	109.117
585	277.878	109.629
590	277.764	110.116
595	277.650	110.579
600	277.536	111.014

Simulated materials for the SIP (Nomex E Felt #2352 NRB) and RTV-560 were not necessary since both materials were readily available.

TPS thickness. - The TPS thickness variation on the Orbiter upper body is relatively small. Table XIV shows the total required Orbiter TPS thicknesses as a function of  $X_0$  and  $Y_0$  coordinates. The TPS RSI tile thicknesses were obtained by subtracting 0.160 in. (a standard allowance used for the SIP and bondline thicknesses) from these values. A review of the TPS thickness requirements resulted in the choice of 1.041 cm (0.410 in.) for the TPS thickness on the large radius test fixture. This choice results in a tile thickness of 0.64 cm (0.25 in.). It was also decided that a uniform TPS thickness could be used over the entire test fixture since the TPS thickness variation was very small over most of the area included in the test fixture design.

The TPS thickness variation on the Orbiter lower body is considerably larger than that on the upper body. Table XV shows the total required TPS thickness on the lower body as a function of  $X_0$  and  $Y_0$  coordinates. These TPS

TABLE XIV  
TOTAL TPS THICKNESS-ORBITER UPPER BODY

Y <sub>0</sub>	X <sub>0</sub>						
	505	520	540	551.15	560	580	600
30	0.9214	0.4001	0.3817		0.3726	0.3699	0.3695
35	0.9256	0.4259	0.3967		0.3768	0.3700	0.3693
40	0.8003	0.4376	0.4123		0.3824	0.3718	0.3699
45	0.5714	0.4255	0.4243		0.3902	0.3758	0.3725
50	0.3671	0.3879	0.4284		0.4003	0.3840	0.3785
55	0.3600	0.3512	0.4217		0.4126	0.3969	0.3882
60	0.3600	0.3600	0.4102	0.4253	0.4253	0.4128	0.4006
65	0.3600	0.3599	0.4036		0.4342	0.4264	0.4126
70	0.3600	0.3600	0.3978	0.4134	0.4243	0.4320	0.4188
75	0.3600	0.3600	0.3978		0.4156	0.4254	0.4259
80	0.3600	0.3601	0.4006	0.4141	0.4189	0.4222	0.4304
85	0.4715	0.4505	0.4052		0.4310	0.4339	0.4447
90	0.5300	0.4953	0.4690		0.4496	0.4609	0.4768
95	0.6263	0.5662	0.5209		0.5007	0.5021	0.5295
100		0.7033	0.6090		0.5703	0.5681	0.6100

- Notes: (1) These values obtained from Rockwell International.
- (2) TPS thickness values are in inches.
- (3) Subtract 0.160 from numbers in table to obtain RSI thickness.

thicknesses vary from a minimum of 3.58 cm (1.41 in.) to a maximum of 7.52 cm (2.96 in.). As a result of discussions between MDAC and NASA-JSC, a decision was made to approximate the TPS surface contours on the small radius test fixture with areas of constant thickness except over the antenna where the thickness would be contoured. This approach was necessary in order to keep the number of different tile thicknesses to a reasonably small number during fabrication. In areas where the thickness variation is small, a thickness step size of 4.06 mm (.160 in.) was selected. In areas outboard of butto line Y<sub>0</sub> = 80, where the thickness increased approximately 6 mm (1/4 in.) for every 15 cm (6 in.), a thickness step size of 6.6 mm (0.26 in.) was selected. This approach results in a TPS surface within +2.03 mm (+0.080 in.) and +3.30 mm (+0.130 in.) of the true surface.

Surface wave considerations: The material thickness for the onset of surface waves propagation (reference 3) is given by

$$\frac{d}{\lambda_0} = \frac{n}{4\sqrt{\epsilon_r - 1}} \quad n = 0, 1, 2, 3,$$

TABLE XV  
TOTAL TPS THICKNESS-ORBITER LOWER BODY

Y <sub>0</sub>	X <sub>0</sub>						
	505	525	545	556	565	585	600
30	1.7198	1.7029	1.6831		1.6750	1.6676	1.6496
35	1.7247	1.6960	1.6638		1.6480	1.6370	1.6172
40	1.7318	1.6893	1.6430		1.6187	1.6040	1.5828
45	1.7417	1.6836	1.6217		1.5881	1.5698	1.5473
50	1.7556	1.6797	1.6007		1.5573	1.5356	1.5121
55	1.7745	1.6790	1.5813		1.5277	1.5028	1.4790
60	1.8004	1.6830	1.5654		1.5012	1.4735	1.4499
65	1.8356	1.6940	1.5552		1.4803	1.4501	1.4274
70	1.8836	1.7153	1.5540		1.4682	1.4359	1.4147
75	1.9497	1.7517	1.5667		1.4694	1.4351	1.4159
80	2.0418	1.8106	1.6000		1.4904	1.4536	1.4365
82	2.0886	1.8428	1.6214	1.5458	1.5064	1.4680	1.4516
84	2.1426	1.8814	1.6486	1.5693	1.5276	1.4874	1.4716
86	2.2051	1.9277	1.6829	1.5993	1.5553	1.5125	1.4970
88	2.2777	1.9832	1.7255	1.6372	1.5904	1.5443	1.5285
90	2.3623	2.0500	1.7784	1.6845	1.6343	1.5836	1.5670
92	2.4612	2.1306	1.8435	1.7431	1.6886	1.6317	1.6136
94	2.5763	2.2280	1.9235	1.8153	1.7555	1.6902	1.6691
96	2.7080	2.3462	2.0221	1.9041	1.8372	1.7605	1.7354
98		2.4895	2.1437	2.0131	1.9371	1.8447	1.8127
99.8	2.96						
100		2.6618	2.2937	2.1469	2.0588	1.9453	1.9039
102			2.478	2.3108	2.2067	2.0647	2.0102
103.1		2.96					
104						2.2059	2.1337
105			2.82		2.49	2.29	2.20
106.4			2.96				
107					2.71	2.46	2.35
109						2.66	2.52
109.7					2.96		
111						2.86	2.71
113						2.96	2.88
115.4							2.96

- Notes: (1) These values obtained from Rockwell International.  
 (2) TPS thickness values are in inches.  
 (3) Subtract 0.160 from numbers in table to obtain RSI thickness.

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where

- d = thickness of dielectric coating
- $\lambda_0$  = free-space wavelength
- $\epsilon_r$  = relative dielectric constant
- n = order of surface wave mode.

Even  $TM_n$  modes (i.e., polarization perpendicular to ground plane) and odd  $TE_n$  modes (i.e., polarization parallel to ground plane) can exist. Since the antennas to be tested on the respective test fixtures are circularly polarized, both perpendicular and parallel polarization must be considered. From the above expression, it is obvious that surface waves for polarization perpendicular to the ground plane ( $TM_0$  mode) can always exist. However, for polarization parallel to the ground plane the thickness required for surface wave propagation is finite and is determined by the dielectric constant of the ground plane cover material.

The TPS thickness for the onset of the  $TE_1$  surface wave mode is 0.645 wavelengths, assuming a TPS dielectric constant of 1.15 and an operating frequency of 2252.5 MHz (center transmit frequency specified by RFP). The TPS thickness in wavelengths ( $d/\lambda$ ) on the small radius test fixture ranges from 0.269 to 0.565, and therefore, only the  $TM_0$  surface wave mode can be excited. Figure 46 shows a plot of the  $TM_0$  propagation velocity relative to free-space,  $c/v$  (also propagation phase constant relative to free-space,  $\beta/k$ ), for a lossless dielectric ( $\epsilon_r = 1.15$ ) of thickness  $d/\lambda$  on a lossless ground plane. Since the SIP dielectric constant ( $\epsilon_r = 1.25$  (reference 2)) is only slightly greater than that of the RSI and the SIP layer is very thin (4.06 mm (0.160 in.)), the effect of the SIP on the  $c/v$  ratio is negligible. The propagation velocity and phase constant of free-space are represented by  $c$  (speed of light) and  $k$  ( $2\pi/\lambda$ ), respectively. Using figure 46, it can be shown that the surface wave propagation velocity will be within 0.2% of that expected for a true surface contour when the steps in TPS tile thickness are 6.6 mm (0.26 in.) or less. Therefore, the approach described above for approximating the TPS thickness requirements on the small radius test fixture is considered technically sound.

Edge effects. - A microwave absorber was added to the test fixture edges (figure 47) to minimize the effects of surface waves and skin currents which radiate from sharp discontinuities and interact with the basic antenna radiation pattern. The absorber selected is Emerson & Cuming AN 77 which has good absorption characteristics (reflectivity  $\leq$  20 dB) down to frequencies of 1.2 GHz.

This technique has been used by MDAC in the past with good results. Figure 48 shows the patterns of a flat spiral antenna measured at 3.0 GHz on a ground plane 56 cm (26 in.) in diameter (which is about 6.6 wavelengths in diameter). The pattern with the polarization perpendicular to the ground plane shows significant ripple due to the ground plane edge. A change in ground plane size changes the frequency of ripple. The pattern with the polarization parallel to the ground plane shows no significant edge effect. By adding a

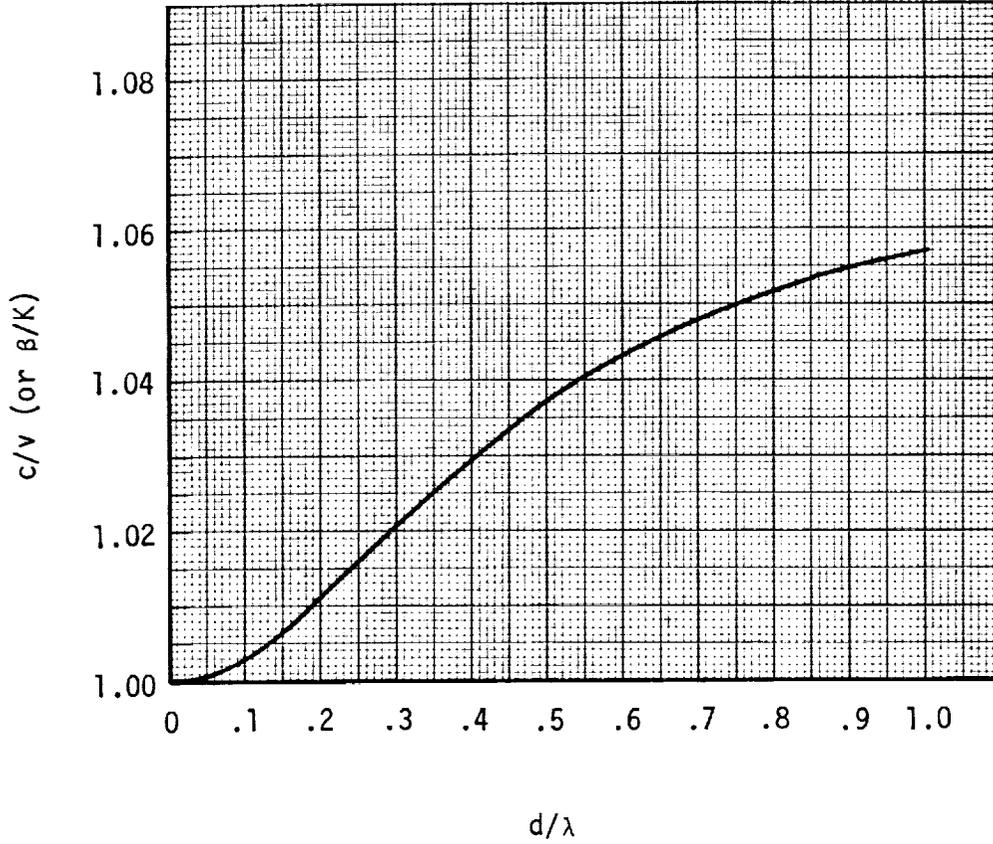


FIGURE 46 PROPAGATION VELOCITY RATIO ( $\epsilon_r = 1.15$ )

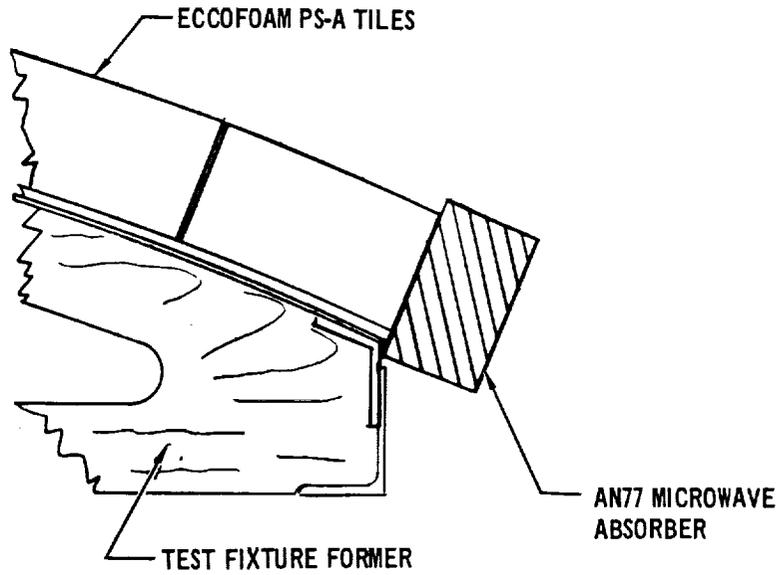


FIGURE 47 MICROWAVE ABSORBER CONFIGURATION FOR MINIMIZING EDGE RADIATION

(a) Polarization Parallel to Ground Plane

(b) Polarization Perpendicular to Ground Plane

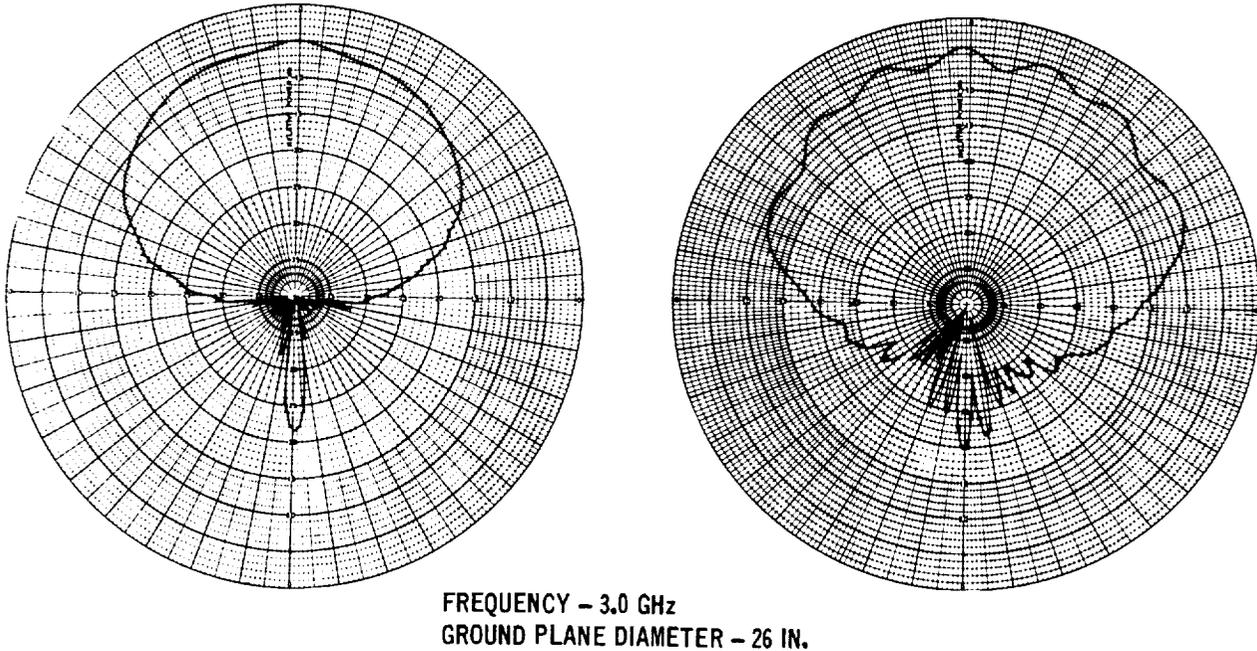


FIGURE 48 EFFECTS OF GROUND PLANE EDGE ON S-BAND SPIRAL ANTENNA PATTERN

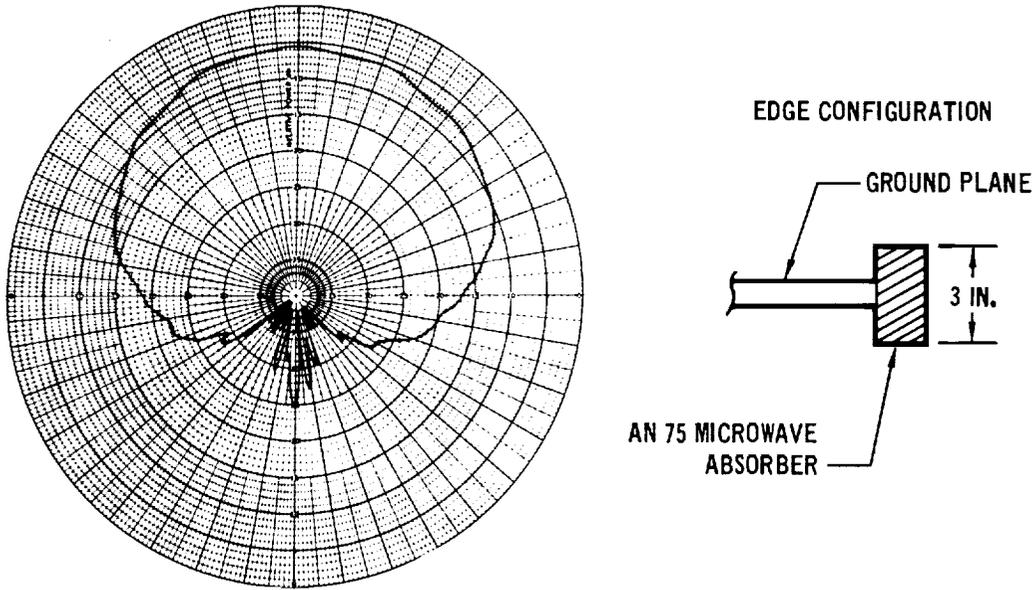
strip of Emerson & Cuming AN 75 microwave absorber 3 in. wide around the edges of the ground plane, the pattern with the polarization perpendicular was significantly improved as shown in figure 49.

Mounting provisions. - The mounting approach selected was one in which arbor plates are attached to the forward and aft ends of the test fixtures (figure 50) to permit rotation in rotisserie fashion. The extension, which spans between the end frame and next inboard frame, eliminates bending loads on the end frame and, thus, ensures adequate support and stiffness during testing and handling. The arbor plate design mates with the standard Scientific Atlanta Series 5860 and 5870 antenna tower arbor plates.

#### Fabrication

The test fixtures were fabricated per the design approach discussed in the preceding subsection on Design. The fabrication processes for both the large and small radius test fixtures were straightforward and essentially the same with a few exceptions as will be noted.

Test fixture structure. - The test fixture structure consists of 1.27 cm (0.50 in.) thick plywood frames and 3.18 x 2.22 x .32 cm (1-1/4 x 7/8 x 1/8 in.) aluminum "T" stringers as shown in figure 51. The frames were constructed with the aid of full-scale mylar drawings of the developed contour lines which were



POLARIZATION - PERPENDICULAR TO GROUND PLANE  
FREQUENCY - 3.0 GHz  
GROUND PLANE DIAMETER - 26 IN.

FIGURE 49 EFFECTS OF ADDING MICROWAVE ABSORBER TO GROUND PLANE EDGE

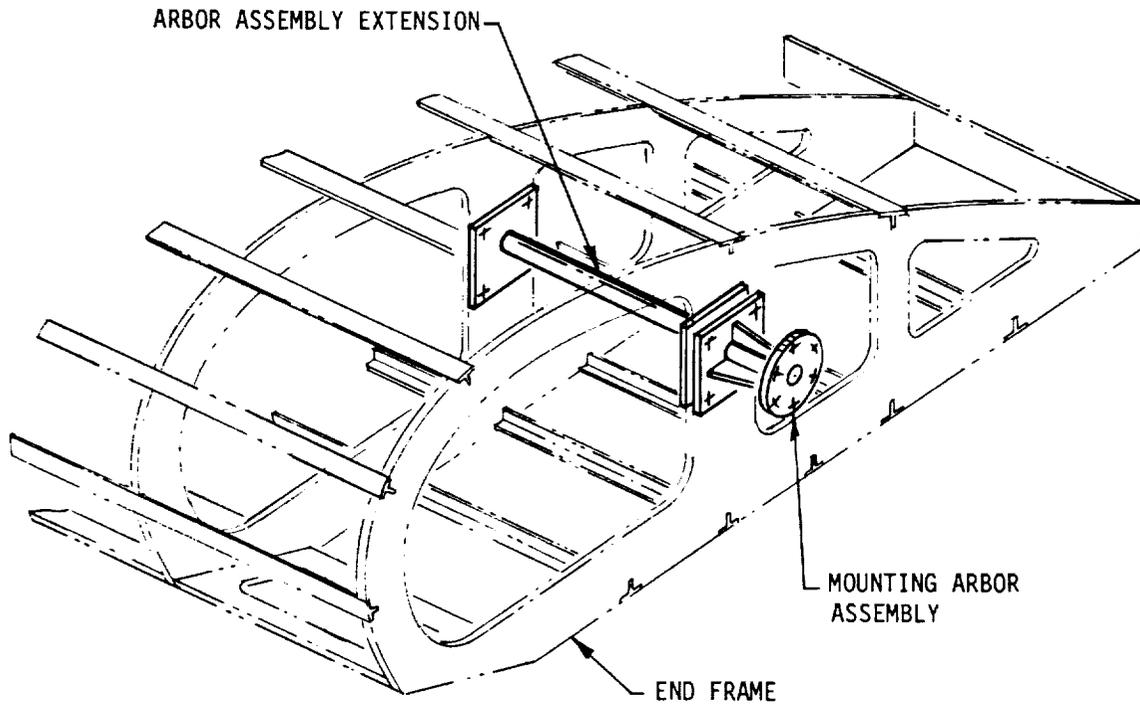


FIGURE 50 MOUNTING ARBOR ASSEMBLY

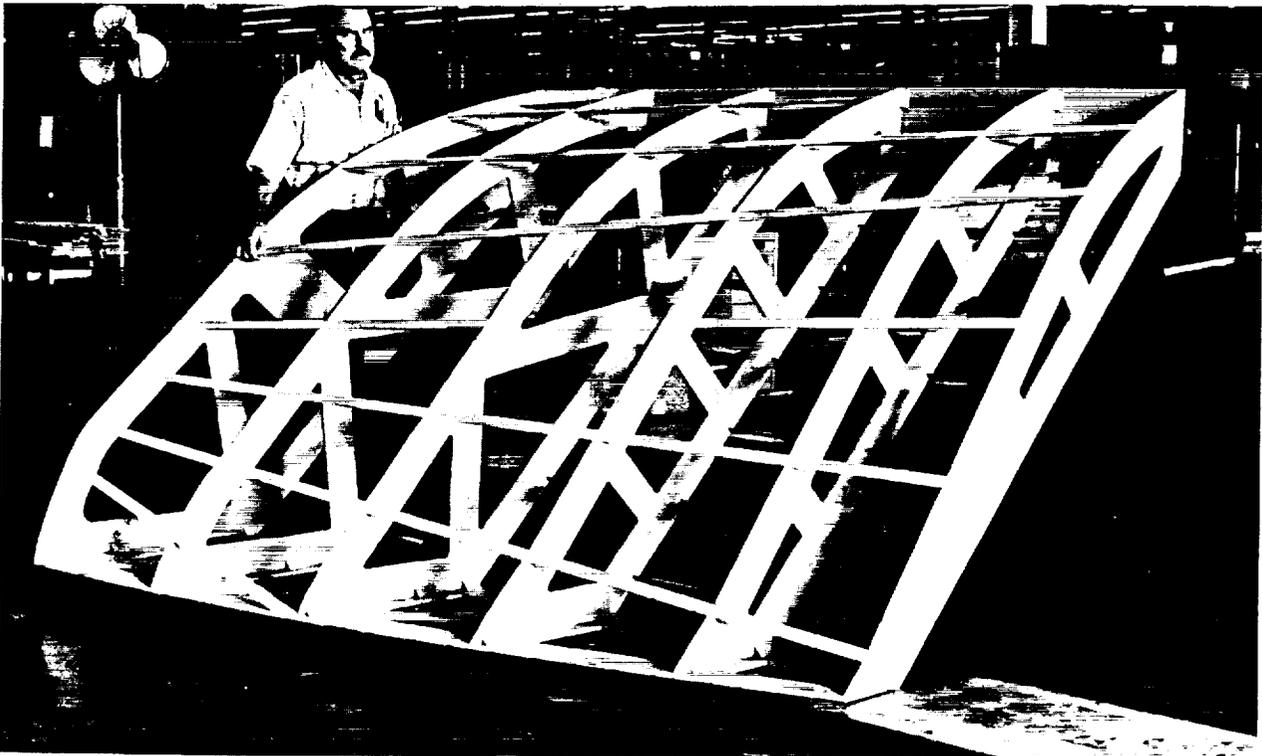


FIGURE 51 LARGE RADIUS TEST FIXTURE STRUCTURE

cut out and temporarily attached with rubber cement. Cut-outs were made in the frames to decrease their weight while maintaining the required strength and stiffness. Seven "T" stringers were arranged along straight line elements to provide uniform support for the skin. Plywood panels were used along the inboard buttline and the outboard waterline to tie the frame ends together and add stiffness to the structure. A 0.813 mm (0.032 in.) aluminum skin was attached to the framework using 3 mm (1/8 in.) brazier head (pull-through type) aluminum alloy Chobert rivets with a head height of 1.27 mm (0.050 in.). Figures 52 and 53 show the large and small radius test fixture structures with the skin attached. (Note: The skin distortions that appear in figure 52 are due to light reflections which greatly exaggerate small irregularities in flat surfaces.) A standard joint sealer (PR-1422, Class B-1/2 polysulfide, commonly used in airplane production) was used along the skin stringer and frame interfaces. This results in a more solid structure which will dampen skin motion and flutter when the test fixture is rotated during testing.

Antenna mounting frame. - Figure 52 also shows the antenna mounting frame. The skin around the antenna mount was backed by a 0.254 cm (0.100 in.) aluminum doubler plate which was contoured to match the skin contours and riveted to the skin near the frames and stringers. A sketch of the antenna mounting frame details is shown in figure 54. The back-side mounting approach was necessary because the TPS is continuous over the antenna and is essentially held in place over the antenna by the SIP. Conducting epoxy was used to avoid machining the complex surface to match the skin contours. Twenty, evenly-spaced, threaded holes, extending through the doubler plate and skin, were

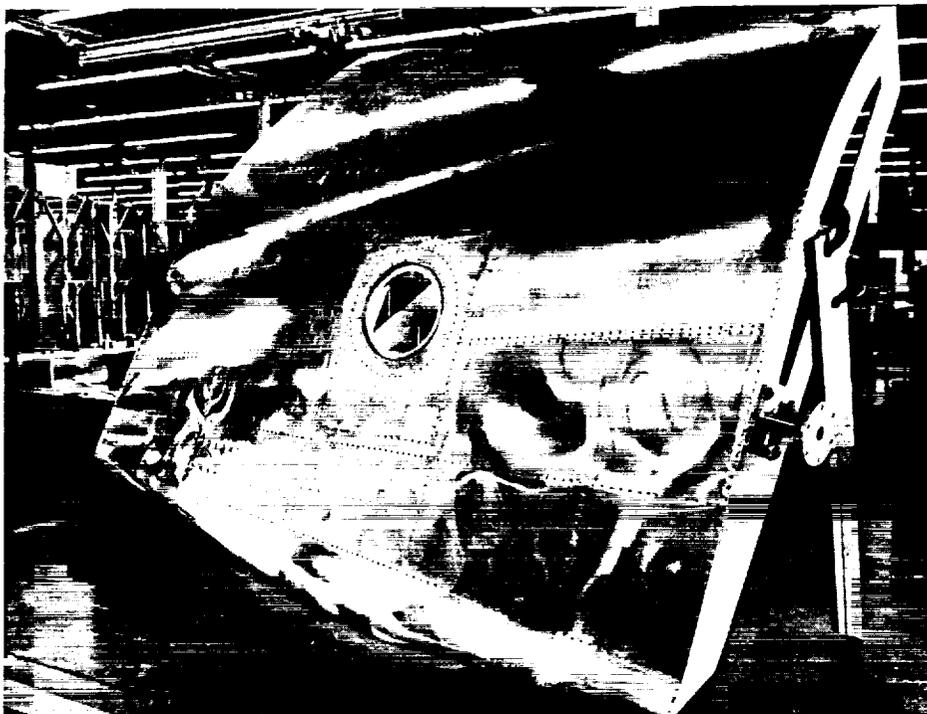


FIGURE 52 LARGE RADIUS TEST FIXTURE STRUCTURE WITH SKIN ATTACHED AND ANTENNA MOUNTING FRAME INSTALLED

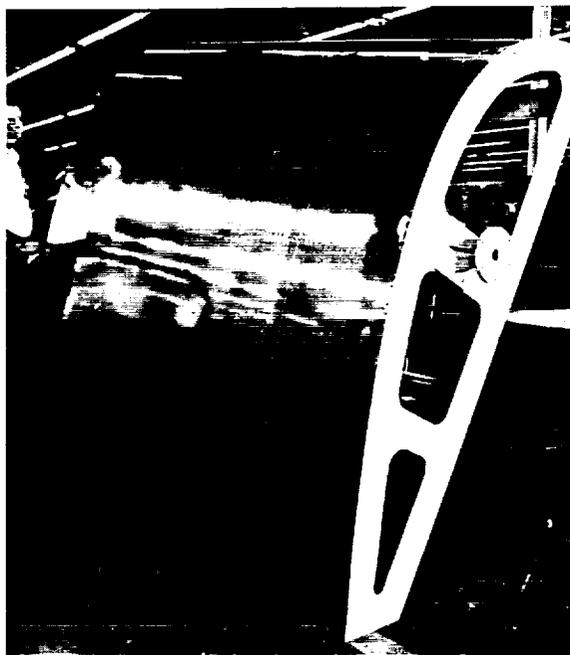


FIGURE 53 SMALL RADIUS TEST FIXTURE STRUCTURE WITH SKIN AND MOUNTING ARBOR ATTACHED

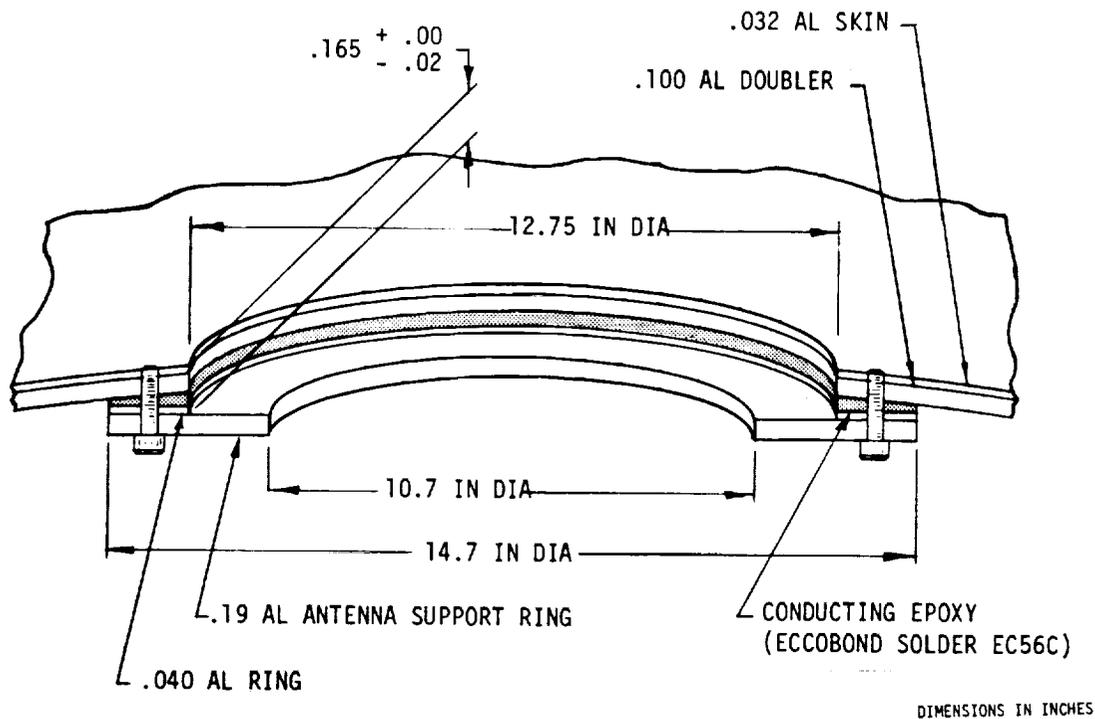


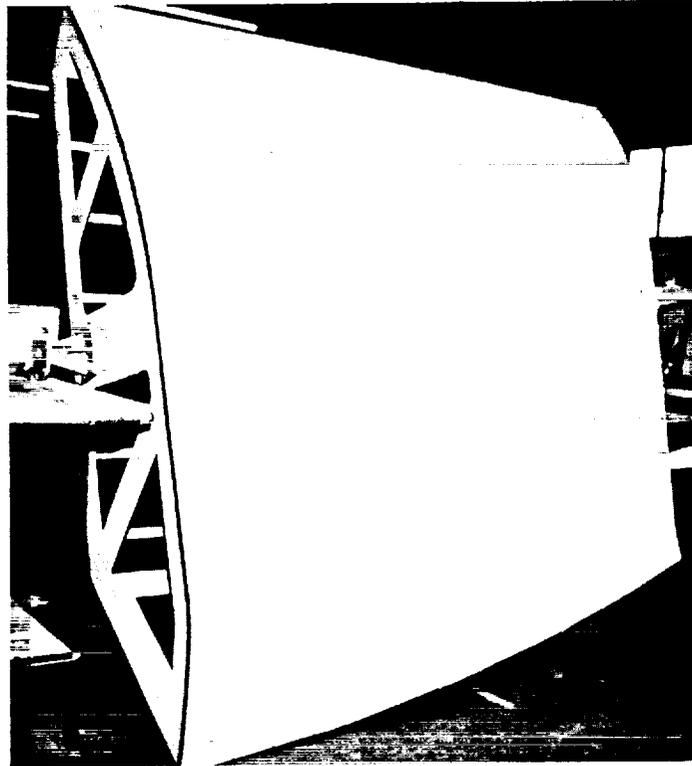
FIGURE 54 ANTENNA MOUNTING DETAILS

provided to attach the antenna support ring and antenna to the test fixture. The attachment screws were cut off so they would not extend above the skin surface and were color coded for convenience.

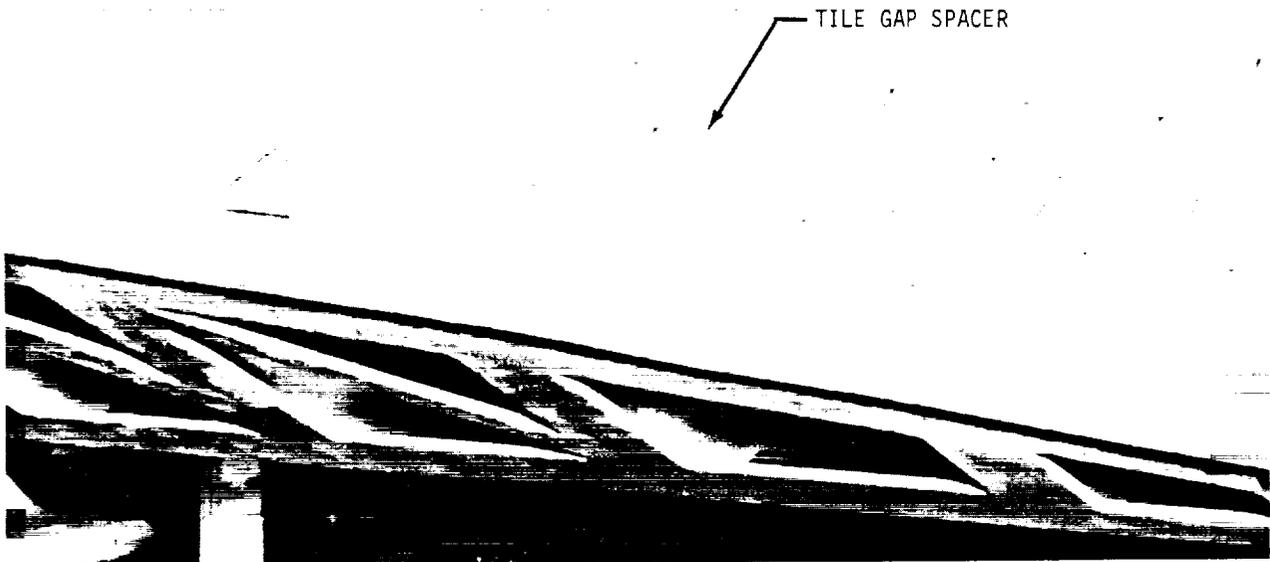
Test fixture interface continuity. - Copper tape 5.1 cm (2.0 in.) wide was attached to the skin and extended over the test fixture edge surfaces about 2.5 cm (1.0 in.) to provide a means for obtaining electrical continuity with an adjacent test fixture without disturbing the simulated TPS.

Simulated TPS attachment. - The SIP was bonded to the test fixture skin with RTV-560 adhesive as shown in figure 55. RTV was applied to the aluminum skin, primed with Dow Corning A4094 primer, and the SIP carefully laid in place. The RTV cannot be applied directly to the SIP because the SIP, which is a porous blanket, readily absorbs the RTV and the bond thickness cannot be controlled. Over the cut-out for the antenna, the SIP was supported by a dummy antenna mounting flange plate, which was formed to the skin surface contours with White Knight autobody putty.

TPS tile fabrication - large radius test fixture. - TPS tiles for the large radius test fixture were fabricated from 43 x 43 x 43 cm (17 x 17 x 17 in.) ECCOFOAM PS-A blocks. The blocks were first cut into 15.24 x 15.24 cm (6.00 x 6.00 in.) logs and then into tiles 6.35 mm (0.25 in.) thick. These tiles were layed out on the test fixture as shown in figure 56. Cardboard strips 1.27 mm (0.050 in.) thick were used to obtain the proper tile spacing. Where necessary, the thin ECCOFOAM tiles were molded to the curvature by using a small amount of heat while forcing them to the contour.



**FIGURE 55 LARGE RADIUS TEST FIXTURE SKIN COVERED WITH NOMEX FELT (SIP)**



**FIGURE 56 ECCOFOAM TILE LAY-OUT BEFORE BONDING**

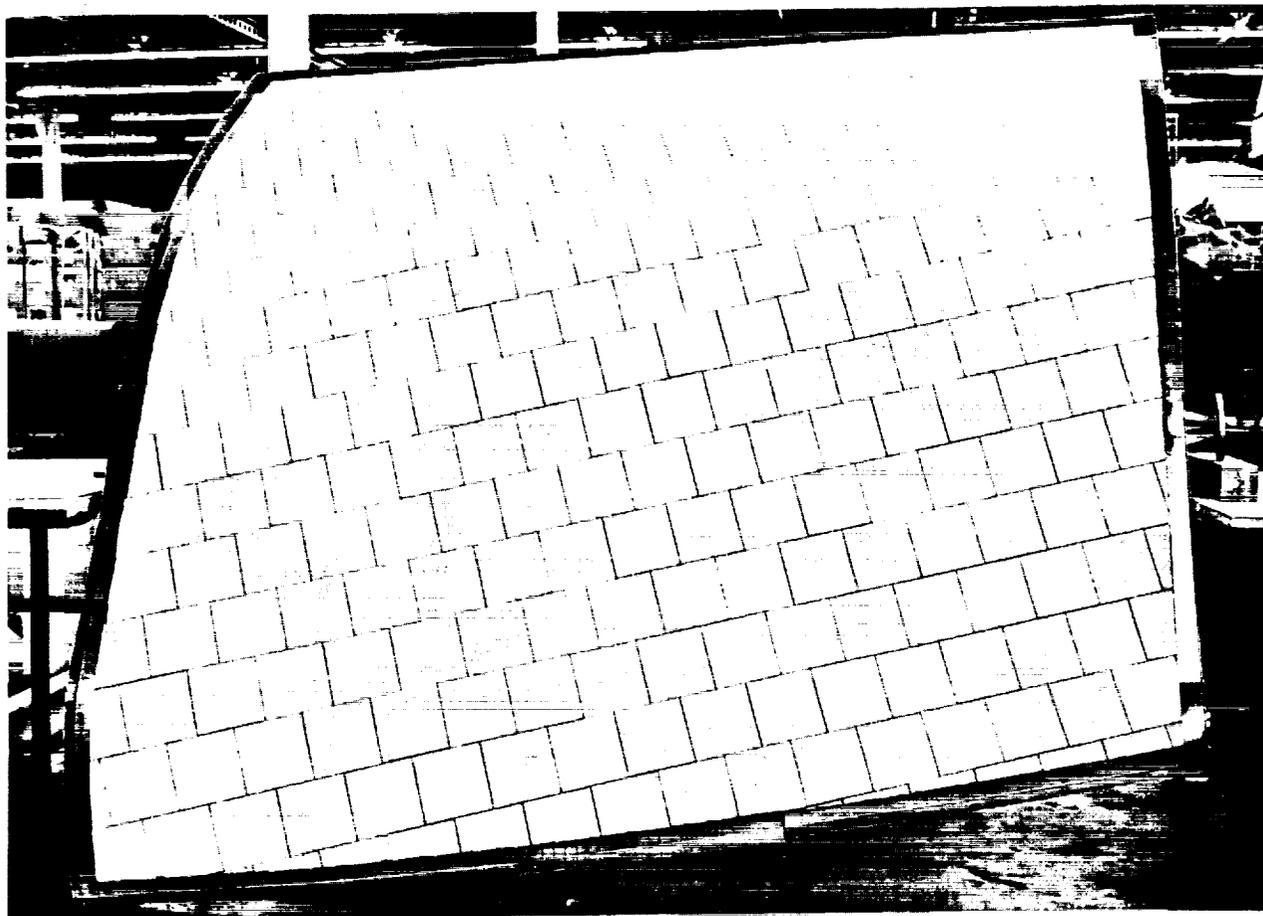
After the tile lay-out was completed, the tile edges were coated with Dow Corning 92-009 silicon. Preliminary tests showed that a Dow Corning A4094 primer applied to the ECCOFOAM PS-A surface would improve the bond strength 2-3 times. However, when the primer was applied to the thin tiles, they curled up as the primer cured and were unusable. Therefore, the tiles on the large radius test fixture were not primed before bonding.

Following cure of the edge coating, the tile surface was coated with RTV and placed on the SIP. A vacuum bag was used to apply a pressure of 6.9 N/m<sup>2</sup> (1 psi) on the tile surface during the cure cycle.

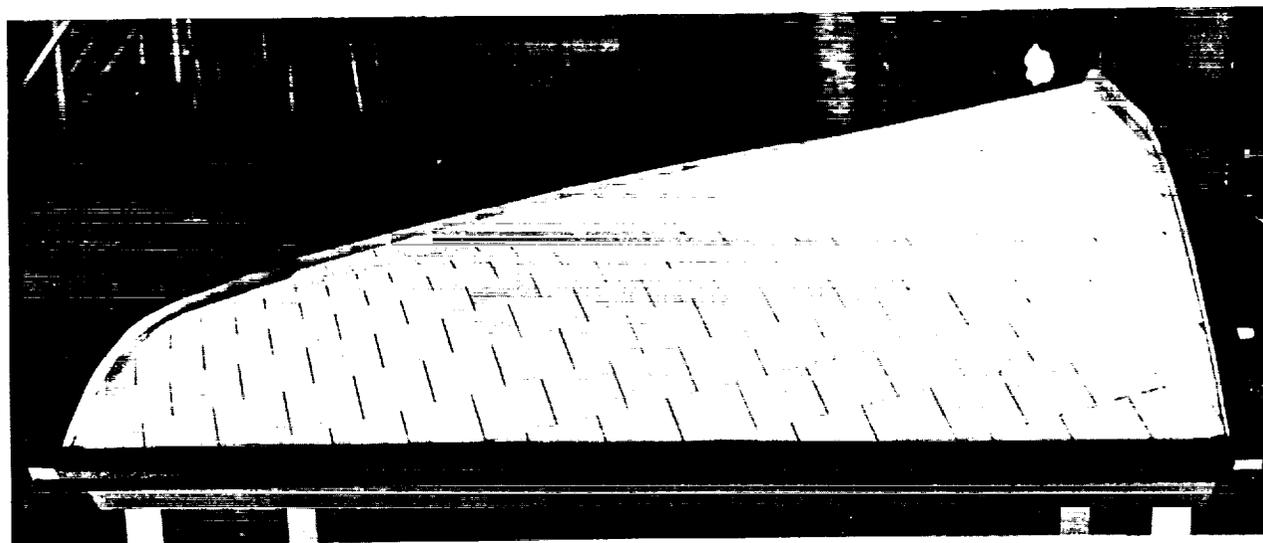
After the RTV cured, the tile surfaces were coated with Dow Corning 92-009 silicon. Three coats were required to obtain a coating thickness of 0.25 to 0.38 mm (0.010 to 0.015 in.). The silicon coating also tended to make the tiles curl slightly although preliminary test samples did not indicate such an effect. Further investigation showed that the curling was most prominent if the ECCOFOAM density was slightly low. Figure 57 shows a top view of the large radius test fixture after completion of bonding and surface coating. Figure 58 shows a photo of the completed large radius test fixture.

TPS tile fabrication - small radius test fixture. - The fabrication of the ECCOFOAM tiles for the small radius test fixture was considerably more difficult because of the variation in tile thickness and the smaller radius of curvature. From about the middle of the test fixture surface and outboard the curvature was such that two sides of the tiles had to be tapered to maintain a reasonable gap width. Further, the tiles thickness in each row decreased from body stations  $X_0 = 505$  to  $X_0 = 600$ . The respective tile thickness as a function of location was determined by developing a flat pattern of the skin surface using the frame lines developed for the simply curved surface. Tile thickness contour plots (figure 59) were made on the flat pattern according to the thickness variation guidelines discussed in the previous section on Design. The respective contour lines bound the TPS thickness selected for each area except the area on the upper surface of the chine (or wing boot). In this region the TPS thickness rapidly decreases to about 2.5 cm (1.0 in.) at the right extremity of the flat pattern. These TPS thickness contours result in tile thicknesses of 3.40, 3.81, 4.22, 4.75, 5.41, 6.07, and 6.73 cm (1.34, 1.50, 1.66, 1.87, 2.13, 2.39, and 2.65 in.). A tile lay-out was made on the flat pattern to identify the tile requirements during fabrication. The tile thickness over the antenna and about 15 cm (6 in.) immediately surrounding it was contoured to conform to actual thickness requirements per table XV. The tile dimensions at the base were held to 15.24 x 15.24 cm (6.00 x 6.00 in.) and the transverse distance was increased as necessary to form a straight gap between tile rows.

The tiles over the antenna were bonded to the test fixture and then the surface was shaped to the required contour. This served as a base for the tile lay-out. The remaining tiles were fabricated and bonded to the SIP one row at a time. The tile edges were coated with silicon prior to bonding as discussed previously.



**FIGURE 57 LARGE RADIUS TEST AFTER COMPLETION OF BONDING AND SURFACE COATING**



**FIGURE 58 LARGE RADIUS ELECTRICAL TEST FIXTURE**



FIGURE 59 TPS THICKNESS CONTOURS - SMALL RADIUS TEST FIXTURE

A thin coat of RTV-560 on the tile surface prior to bonding proved to be an excellent method of priming the tiles. After the tile was coated with RTV for bonding and put into place, force was applied to the tile surface. As before, cardboard gap spacers were used to maintain the proper gap spacing.

After the RTV cure cycle, when the gap spacers were removed it was observed that the gap spacing changed. This was attributed to the loading on the SIP during bonding. When the SIP is compressed, the expansion direction is not necessarily in the same direction as the applied compressive load. Therefore, when the load is removed, the SIP may spring back to its original shape with some sidewise movement. Figure 60 shows the small radius test fixture before all the tiles have been bonded in place. In some areas a variation in gap width can be seen. Figure 61 shows a photo of the completed small radius test fixture. The tile thickness contours can be seen.

Microwave absorber attachment. - Following the tile bonding and surface coating tasks, the Emerson & Cuming AN 77 microwave absorber was attached around the test fixture edges by means of a simple Z bracket as shown in figure 62. The absorber was bonded to the brackets and can be removed by removing the screws holding the brackets in place.

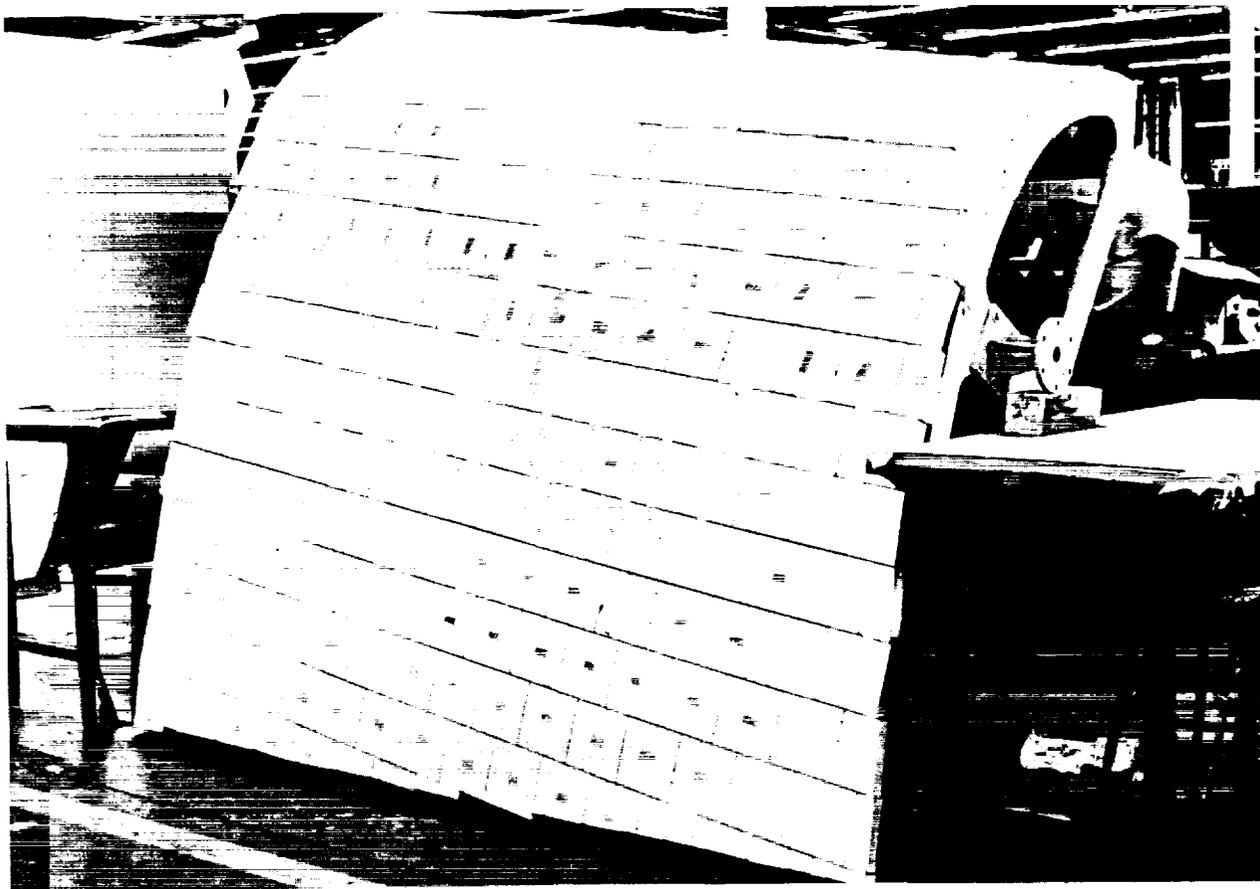


FIGURE 60 SMALL RADIUS TEST FIXTURE DURING TILE BONDING

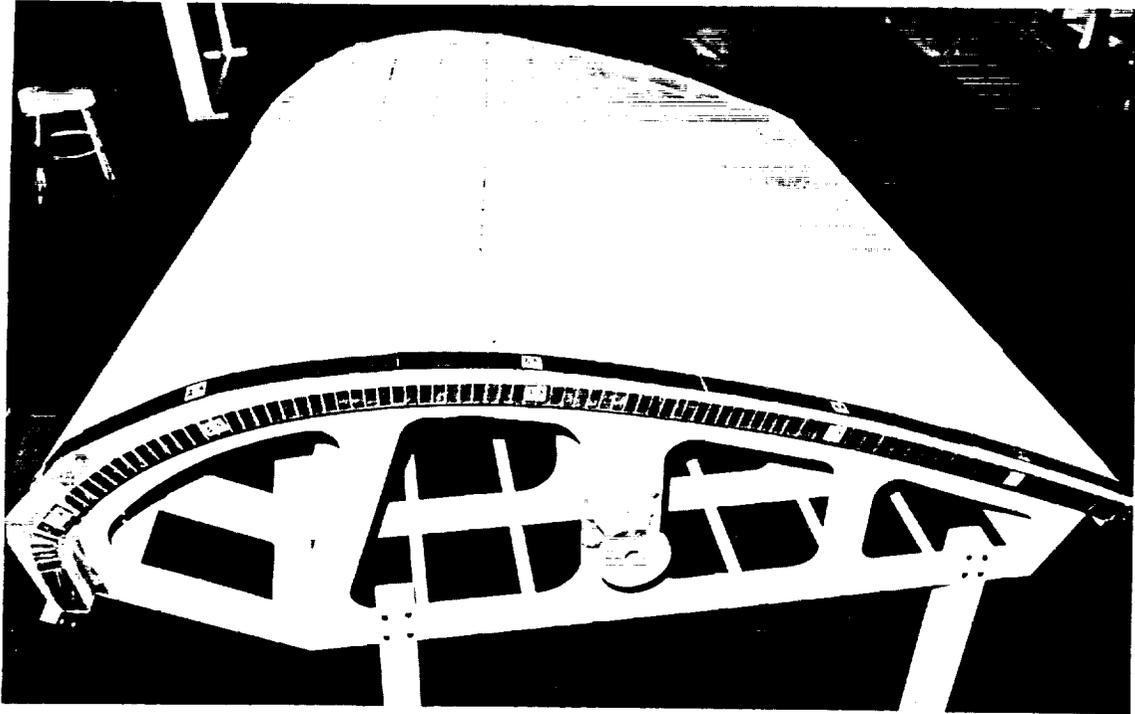


FIGURE 61 SMALL RADIUS ELECTRICAL TEST FIXTURE

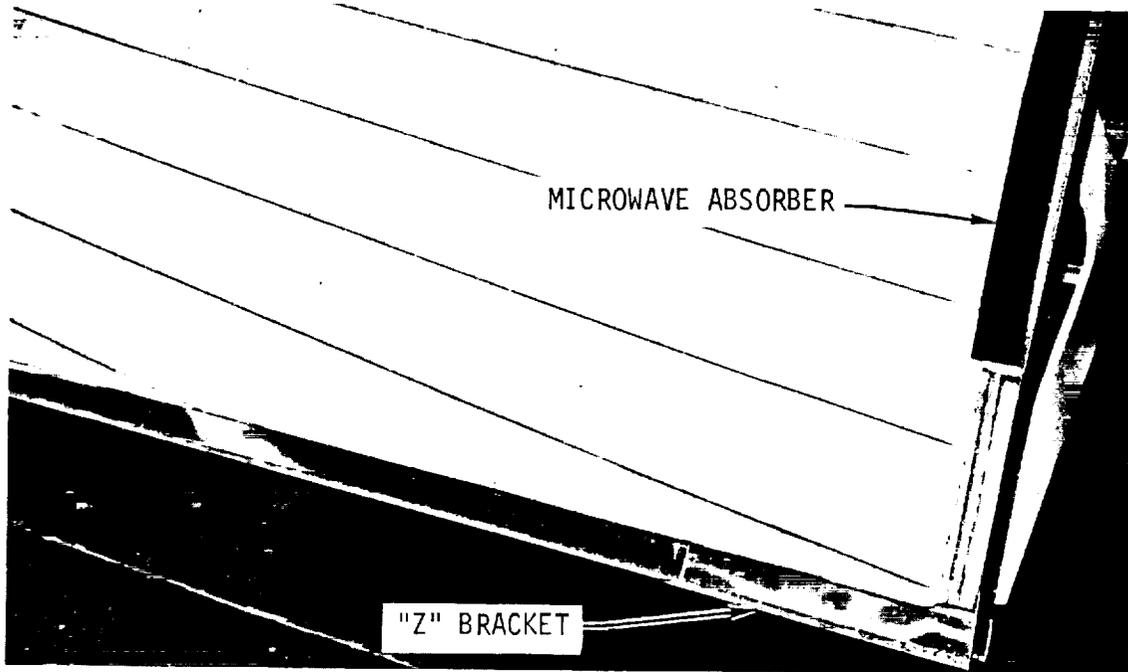


FIGURE 62 MICROWAVE ABSORBER ATTACHMENT CONFIGURATION

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REFERENCES

1. Gilreath, M. C. and Castellow, S. L., Jr.: High-Temperature Dielectric Properties and Candidate Space Shuttle Thermal-Protection-System and Antenna-Window Materials, NASA TN D-7523, June 1974
2. Stricklen, J. O.: "Shuttle TPS Dielectric Constant Tests," Rockwell International Memo No. 098-31-4057, 26 August 1974
3. Bailey, M. C. and Crowell, W. F.: "Pattern Measurements of Slot Radiators in Dielectric-Coated Metal Plates," IEEE Trans. Antennas and Propagation (Communications), Vol. AP-15, pp. 824-826, November 1967

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APPENDIX A

MATERIAL PROPERTIES

The material properties given below were used in the analyses performed in this program.

(1) REUSABLE SURFACE INSULATION (HRSI/LRSI)

Temperature (°F)	Thermal Conductivity (BTU/ft-hr-F)					Specific Heat (BTU/lb-F)
	P = 0.21 psf (P ~ 0.076 torr)	P = 2.12 psf (P ~ 0.76 torr)	P = 21.16 psf (P ~ 7.6 torr)	P = 211.6 psf (P ~ 76 torr)	P = 2116 psf (P ~ 760 torr)	
-250.	0.0050	0.0075	0.0150	0.0216	0.0233	0.070
-150.	-	-	-	-	-	0.105
0.	0.0075	0.0100	0.0183	0.0250	0.0275	0.150
250.	0.0092	0.0125	0.0225	0.0316	0.0341	0.210
500.	0.0125	0.0167	0.0276	0.0400	0.0433	0.252
750.	0.0175	0.0216	0.0325	0.0492	0.0534	0.275
1000.	0.0233	0.0275	0.0392	0.0600	0.0658	0.288
1250.	0.0308	0.0350	0.0492	0.0725	0.0782	0.296
1500.	0.0416	0.0459	0.0617	0.0875	0.0942	0.300
1750.	0.0567	0.0610	0.0767	0.1060	0.1130	0.302
2000.	0.0734	0.0782	0.0942	0.1270	0.1360	0.303
2300.	0.0966	0.1020	0.1160	0.1550	0.1670	0.303

Density = 9.0 lb<sub>m</sub>/ft<sup>3</sup>

Emissivity = 0.8

Solar Absorptivity =  $\begin{cases} 0.8 \text{ HRSI} \\ 0.32 \text{ LRSI} \end{cases}$

(2) STRAIN ISOLATOR PAD (SIP)

Temperature (°F)	Thermal Conductivity (BTU/ft-hr-F)				
	P = 0.21 psf (0.076 torr)	P = 2.12 psf (0.76 torr)	P = 21.16 psf (7.6 torr)	P = 211.6 psf (76 torr)	P = 2116 psf (760 torr)
-282	0.0050	0.0069	0.0090	0.0094	0.0094
-162	0.0050	0.0083	0.0130	0.0140	0.0140
92	0.0050	0.0120	0.0210	0.0230	0.0240
396	0.0090	0.0170	0.0300	0.0370	0.0380

Temperature (°F)	Specific Heat (BTU/lb <sub>m</sub> -F)
-238	0.105
-148	0.124
-58	0.158
32	0.208
122	0.277
212	0.353
302	0.462
392	0.564

Density = 5.4 lb<sub>m</sub>/ft<sup>3</sup>

(3) INTERNAL INSULATION (TG-15000)

Temperature (°F)	Thermal Conductivity (BTU/ft-hr-F)				
	P = 0.0278 psf (P ~ 0.01 torr)	P = 0.278 psf (P ~ 0.1 torr)	P = 2.78 psf (P ~ 1.0 torr)	P = 27.8 psf (P ~ 10 torr)	P = 2116 psf (P ~ 760 torr)
-300	0.0005	0.0006	0.0016	0.0035	0.0048
-200	0.0009	0.0012	0.0028	0.0061	0.0077
-100	0.0015	0.0018	0.0041	0.0088	0.0105
0	0.0021	0.0026	0.0056	0.0117	0.0138
100	0.0030	0.0038	0.0078	0.0150	0.0184
200	0.0042	0.0053	0.0107	0.0196	0.0245
300	0.0058	0.0078	0.0144	0.0250	0.0300
400	0.0078	0.0098	0.0182	0.0296	0.0354
500	0.0102	0.0127	0.0217	0.0342	0.0400
600	0.0127	0.0158	0.0259	0.0384	0.0450
700	0.0154	0.0192	0.0296	0.0421	0.0480

Density = 2.0 lb<sub>m</sub>/ft<sup>3</sup>  
Specific Heat = 0.2 BTU/lb<sub>m</sub>-°F  
Emissivity = 0.2

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(4) MISCELLANEOUS

Material	Density (lb <sub>m</sub> /ft <sup>3</sup> )	Specific Heat (BTU/lb <sub>m</sub> -F)	Thermal Conductivity (BTU/ft-hr-F)	Emittance																												
RTV-560	88.0	0.35	0.18																													
RTV-511	73.7	0.33	<table border="0"> <tr> <td>°F</td> <td>k</td> </tr> <tr> <td>0</td> <td>0.17</td> </tr> <tr> <td>77</td> <td>0.17</td> </tr> <tr> <td>225</td> <td>0.15</td> </tr> <tr> <td>400</td> <td>0.15</td> </tr> </table>	°F	k	0	0.17	77	0.17	225	0.15	400	0.15																			
°F	k																															
0	0.17																															
77	0.17																															
225	0.15																															
400	0.15																															
Polyimide Fiberglass	112.4	0.40	0.184																													
Eccosphere Insulation	5.7	0.19	0.058																													
MLI	0.001	0.001	<table border="0"> <tr> <td>°F</td> <td>k</td> </tr> <tr> <td>-460</td> <td>0.0006</td> </tr> <tr> <td>-300</td> <td>0.0012</td> </tr> <tr> <td>-200</td> <td>0.0028</td> </tr> <tr> <td>-100</td> <td>0.0068</td> </tr> <tr> <td>0</td> <td>0.016</td> </tr> <tr> <td>100</td> <td>0.026</td> </tr> <tr> <td>200</td> <td>0.040</td> </tr> <tr> <td>300</td> <td>0.060</td> </tr> </table>	°F	k	-460	0.0006	-300	0.0012	-200	0.0028	-100	0.0068	0	0.016	100	0.026	200	0.040	300	0.060	0.2										
°F	k																															
-460	0.0006																															
-300	0.0012																															
-200	0.0028																															
-100	0.0068																															
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2024-T8XX A1	178.	<table border="0"> <tr> <td>°F</td> <td>cp</td> </tr> <tr> <td>-200</td> <td>0.147</td> </tr> <tr> <td>0</td> <td>0.195</td> </tr> <tr> <td>200</td> <td>0.216</td> </tr> <tr> <td>300</td> <td>0.224</td> </tr> <tr> <td>400</td> <td>0.233</td> </tr> <tr> <td>600</td> <td>0.250</td> </tr> </table>	°F	cp	-200	0.147	0	0.195	200	0.216	300	0.224	400	0.233	600	0.250	<table border="0"> <tr> <td>°F</td> <td>k</td> </tr> <tr> <td>-200</td> <td>69.6</td> </tr> <tr> <td>0</td> <td>84.0</td> </tr> <tr> <td>200</td> <td>95.0</td> </tr> <tr> <td>300</td> <td>99.0</td> </tr> <tr> <td>400</td> <td>102.5</td> </tr> <tr> <td>600</td> <td>104.5</td> </tr> </table>	°F	k	-200	69.6	0	84.0	200	95.0	300	99.0	400	102.5	600	104.5	0.85
°F	cp																															
-200	0.147																															
0	0.195																															
200	0.216																															
300	0.224																															
400	0.233																															
600	0.250																															
°F	k																															
-200	69.6																															
0	84.0																															
200	95.0																															
300	99.0																															
400	102.5																															
600	104.5																															
6061-T6 A1	169.3	Same as 2024-T8XX	96.8	0.8 0.9*																												

\*Dust cover outside.

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APPENDIX B

ANTENNA THERMAL MODEL NODE LOCATIONS

1	Forward Radiating Element Insulation, Bottom
2	Forward Radiating Element Insulation, Bottom
3	Forward Radiating Element Ridge, Bottom
4	Forward Radiating Element Insulation, Bottom
5	Forward Radiating Element Ridge, Bottom
6	Forward Radiating Element Insulation, Bottom
7	Lateral Radiating Element Insulation, Bottom
8	Lateral Radiating Element Insulation, Bottom
9	Lateral Radiating Element Ridge, Bottom
10	Lateral Radiating Element Insulation, Bottom
11	Lateral Radiating Element Ridge, Bottom
12	Lateral Radiating Element Insulation, Bottom
13	Center Radiating Element Insulation, Bottom
14	Center Radiating Element Ridge, Bottom
15	Center Radiating Element Insulation, Bottom
16	Center Radiating Element Insulation, Bottom
17	Forward Radiating Element Insulation, Center
18	Forward Radiating Element Insulation, Center
19	Forward Radiating Element Ridge, Center
20	Forward Radiating Element Insulation, Center
21	Forward Radiating Element Ridge, Center
22	Forward Radiating Element Insulation, Center
23	Lateral Radiating Element Insulation, Center
24	Lateral Radiating Element Insulation, Center
25	Lateral Radiating Element Ridge, Center
26	Lateral Radiating Element Insulation, Center
27	Lateral Radiating Element Ridge, Center
28	Lateral Radiating Element Insulation, Center
29	Center Radiating Element Insulation, Center
30	Center Radiating Element Ridge, Center
31	Center Radiating Element Insulation, Center
32	Center Radiating Element Insulation, Center
33	Forward Radiating Element Insulation, Top
34	Forward Radiating Element Insulation, Top
35	Forward Radiating Element Ridge, Top
36	Forward Radiating Element Insulation, Top
37	Forward Radiating Element Ridge, Top
38	Forward Radiating Element Insulation, Top
39	Lateral Radiating Element Insulation, Top
40	Lateral Radiating Element Insulation, Top
41	Lateral Radiating Element Ridge, Top
42	Lateral Radiating Element Insulation, Top
43	Lateral Radiating Element Ridge, Top
44	Lateral Radiating Element Insulation, Top
45	Center Radiating Element Insulation, Top
46	Center Radiating Element Ridge, Top

APPENDIX B

(Continued)

47	Center Radiating Element Insulation, Top
48	Center Radiating Element Insulation, Top
49, 50	Forward Radiating Element Canister, Bottom
51, 52	Lateral Radiating Element Canister, Bottom
53	Center Radiating Element Canister, Bottom
54, 55	Forward Radiating Element Canister Side, Bottom
56, 57	Lateral Radiating Element Canister Side, Bottom
58	Center Radiating Element Canister Side, Bottom
59, 60	Forward Radiating Element Canister Side, Center
61, 62	Lateral Radiating Element Canister Side, Center
63	Center Radiating Element Canister Side, Center
64, 65	Forward Radiating Element Canister Side, Top
66, 67	Lateral Radiating Element Canister Side, Top
68	Center Radiating Element Canister Side, Top
69	Polymide Antenna Cover
70-79	Antenna Mounting Plate
80-84	Dust Cover Support Plate
85-88	TPS Bondline Over Antenna
89-103	Radiosity, Radiating Element Canisters
104-112	Forward Radiating Element Hybrid
113, 114	Forward Radiating Element Canister Stud
115, 116	Forward Radiating Element Canister Legs
117-123	Lateral Radiating Element Hybrid
124, 125	Lateral Radiating Element Canister Stud
126, 127	Lateral Radiating Element Canister Legs
128-133	Center Radiating Element Hybrid
134	Center Radiating Element Canister Stud
135	Center Radiating Element Canister Leg
136-141	Dust Cover Sides
142-145	Dust Cover Flange
146-160	Aluminum Skin and Stringers
161-172	TPS Bondline Away From Antenna
173-182	Circuit Board, Outboard
183-190	Radiosity, Dust Cover Sides and Dust Cover Flange
191-198	Radiosity, Circuit Board, Top
199, 200	Circuit Board Sides
201-208	Radiosity, Antenna Mounting Plate
209-224	RSI Inboard, Layer 1
225-234	Circuit Board, Inboard
235	Dust Cover Top
236, 237	Radiosity, Circuit Board Side
238	Radiosity, Dust Cover Top, Inside
239-248	Radiosity, Circuit Board, Inboard
249-264	RSI Layer 2
265-271	TG-15000
272-275	MLI
276	Cabin Wall

APPENDIX B

(Continued)

277-292	RSI Layer 3
293-308	RSI Layer 4
309-324	RSI Surface, Layer 5

1. Refer to figure 4 for component locations.
2. Refer to figures 5 thru 9 for node locations.

**DEVELOPMENT OF S-BAND ANTENNA INTERFACE DESIGN**

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