Project Title: High Temperature Underground Thermal Storage of Solar Energy

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Project Goals: The objective of this project is to establish the feasibility of high temperature underground thermal storage of energy and arrive at a practical system design.

Project Status: Results to date indicate that salt cavern storage of hot oil is both technically and economically feasible as a method of storing huge quantities of heat at relatively low cost. One particular system identified in this study utilizes a gravel filled cavern leached within a salt dome. Thermal losses are shown to be less than one-percent of cyclically-transferred heat. A system like this having a 40 MWt transfer rate capability and over eight hours of storage capacity is shown to cost about $13.50 per KWhr.

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High Temperature Underground Thermal Storage of Solar Energy

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Foundation of Texas from the Solar Energy Research Institute,
Contract # XP-9-8319-1.

Objectives: This report documents the second year of a feasibility
study of underground storage of solar energy as sensible heat. This effort
addresses storage temperatures high enough to utilize conventional steam-
electric power generation on the recovery cycle. The method of storage now
under evaluation utilizes cavern storage of heat transfer oil at temperatures
up to 650°F in leached caverns within salt domes. A study of aquifer storage
of hot water at these temperatures was discontinued when it became apparent
that such storage would encounter major problems from mineral (silica) solution
and requirements for down-hole pumps for the recovery cycle. Research and
development efforts have been focused on the following technical problems:

a) Thermal losses
b) Cavern stability
c) Cavern construction
d) Well designs
e) heat exchanger interfacing
f) economics for cavern storage systems.

Conclusions. Studies indicate that salt cavern storage of hot oil will
be both technically and economically practical as a method of solar energy storage
for electric power generation. The best system identified in this study is a
gravel filled cavern using at least one input and one output well, operated
in a thermocline mode with injection and retrieval on a diurnal cycle. Thermal
losses should be less than one percent of cyclicly-transferred heat. The
gravel filling would act as a heat storage medium and as a stabilizer against
cavern deformation due to plastic flow of salt. (See Figure 1.).

During the second contract year it has been shown that such a system
can be built using existing technology and available materials. In particular,
the design and operation of such a system for interfacing to a 10 MW_e central
receiver like the one being built at Barstow, California has been evaluated.
A cavern storage system could be built for about $4 million having a 10 MW_e
Transfer rate capability and 8+ hours of storage capacity. Storage would be at about 650°F and cost about $13.50 per kWh. This compares favorably with DOE objectives, but larger cavern storage systems would be more cost effective with costs estimated at a low $7.50 per kWh. Thus, cavern storage would be preferred to above-ground storage where it is geologically feasible. A review and summary of the various studies carried out during the past year isolate and explain primary conclusions.

Figure 1. Schematic of gravel filled salt cavern for thermocline storage of heat using heat transfer oil.

Thermal Losses from Cavern Storage Systems. In the first year of the five-year study, we have reported results on thermal losses, calculated with a computer simulator assuming a cavern with perfect mixing (homogeneous temperature inside), for cyclic operating conditions. Those studies showed that for daily cycles of a hot oil storage cavern---8 hours injection and 16 hours retrieval---thermal losses would decline rapidly from a moderately high
value to less than one percent of the transferred energy by the end of one year of operation. This model has also been used to study effects of non-cyclic operation that would result with shut-down of the solar collector in cloudy weather. These studies indicated that losses would not be prohibitive for shut-down periods up to several days if the cavern were sufficiently large.

The simulator under study is an oversimplified model, for it assumes perfect mixing of oil within a spherical cavern. The model also describes a storage system consisting of two caverns, a hot cavern and a cold cavern with a nitrogen gas cap in each cavern. The nitrogen gas cap is compressed during fluid injection and the expansion of the gas produces back-flow in the retrieval cycle, thus negating the requirement of a downhole pump. However this system requires caverns at significant depth to sustain required internal pressures and poses problems of mechanical stability.

It now appears that the preferred cavern storage system will be a single gravel filled cavern with two wells operated in a thermocline mode using oil and rocks essentially like the above ground tanks; one well connects to the top of the cavern (the hot well) and one to the lower end of the cavern (the cold well). (See Figure 1.)

Gravel filling in a thermal storage cavern serves three purposes: 1) the gravel is a storage medium for sensible heat and reduces the required oil volume, 2) the gravel restricts thermal convection and stabilizes the thermocline, and 3) the gravel provides mechanical support and rigidity to the cavern to prevent cavern deformation due to creep of the salt, or plastic flow, provided that internal fluid pressure is less than external geostatic pressure.

The perfect mixing model has therefore been put aside and attention devoted to the formulation of computer simulators for the study of fluid movement, heat transfer, and thermal losses in a gravel filled cavern operated as a thermocline with oil. Preliminary results from these models show that the order of magnitude of the thermal loss in this system is not radically different from that in the homogeneous cavern.

Two new simulators are being evaluated; the first is designed to describe in detail the mass and heat flow within the cavern as well as the heat flow within the salt, while the second simulator is designed for accurate description of the heat flow across the cavern boundaries and within the salt
while the mass and heat flow within the cavern are treated by more approximate means. This program is operational and is being used to generate systematic data on cavern operations.

The model has already been used to show that a thermocline cavern storage system for a small scale system (10 MW<sub>e</sub> with eight hours storage would lose about 2.6% of the useful stored heat during one daily cycle after about three months of continuous operation. The loss rate at the end of one year of continuous operation is approximately 1.5%. For systems of larger size (100 MW<sub>e</sub> or more) the long term loss rate would be 1%, or less, of the cyclicly transferred heat.

Cavern Stability Studies. The major problem anticipated for the cavern storage system was cavern deformation due to creep of the salt at high temperatures, or plastic flow. Published experimental data for a wide range of strains, rates of strain and temperatures have been curve-fitted to provide an adequate description of the viscoelastic/plastic properties of halite. This rheological model for halite is given as a general equation of strain rate as a function of strain, differential stress and temperature for uni-axial conditions. For particular constant stress and temperature values it can be integrated numerically to give the total strain as a function of time. For a temperature of 599°F and an expected differential pressure of 300 psi, the integration yields a total strain on an element of halite of 1.8% in 20 years or 2.2% in 50 years. The same element of halite, when subjected to a differential pressure of 600 psi at the same temperature would suffer a total deformation of 8.4% in 20 years or 14% in 50 years. Note that these results do not take into account the rigidity of the gravel pack which would function to reduce these deformations.

Gravel filling will be a greater deterrent to cavern deformation than initially anticipated. It is well known that a container filled with rigid granules in firm contact cannot undergo a shear deformation without expansion of the container volume. Thus, since the pressure of the overburden must be overcome, in order for the cavern to expand, the gravel should give great rigidity to the cavern. A simulator is currently being developed to determine the deformation and rates of deformation experienced by a spherical cavity filled with saturated gravel, the boundary of which is subjected to hydrostatic loading. This simulator is now in the debugging stage. A survey of the literature is also underway to find sufficient data to determine the elastic/creep constants of saturated sands to be used as input to the simulator.
The critical factor for the stabilizing role of the gravel on the cavern appears to be maintenance of fluid pressure within the pore space less than the confining geostatic pressure of the salt. An analytical solution for the pressure distribution within the cavern has been obtained, valid within some simplifying assumptions, which shows that fluid pressure increases required to maintain fluid flow will generally be on the order of a few hundred psi. Such pressures could be tolerated in caverns at reasonable depths. Further study of these flow effects is underway, using both analytical and numerical methods.

**Thermal Storage Fluids.** The optimum fluid to be used in the cavern storage system would minimize replacement costs for a given power from the system. Replacement costs are directly proportional to fluid loss rate due to thermal degradation. Published experimental data for fluid loss rates of some commercially available heat transfer fluids have been examined from this viewpoint. Caloria HT43, SUN 21 and Therminol 66 were considered. Of these it appears that Caloria HT 43 would be most suitable with a loss rate of about 6.25% per year at 600°F. However, more information is required on effects of contact with metals, air, water and salt on rate of degradation.

**Well Designs and Cavern Construction.** A comprehensive review of the literature indicated that existing equipment, materials, and procedures for oil wells can be adopted for construction of wells for the cavern storage system. The main body of the second-year report on this project provides a detailed description of the process of drilling, completing the wells, leaching the caverns, and placing gravel in the caverns.

The major consideration in the well designs is the requirement of adequate cement bond in the installation of the well casing pipe to constrain the pipe against thermal expansion. The thermal expansion of the injection tubing can be accommodated by thermal expansion joints. A rigid, porous silica foam coating on the exterior of this tubing can provide good thermal insulation.

Other major considerations in the design of cavern systems are brine disposal while leaching and the technique of gravel placement.

Detailed casing designs have been formulated for wells of three flow rates, 1500 gpm, 2000 gpm, and 2500 gpm, and two depths, 3000 ft and 5000 ft. Detailed cost estimates account for materials, installation, and supervision. Costs have also been estimated for cavern leaching and gravel filling.
Heat Exchangers and Power Generation. Since a specific solar collector has not yet been identified to interface with a cavern heat storage system, the 10 MW<sub>e</sub> system being built at Barstow, California, was used for evaluation purposes. Using data provided in the McDonnell Douglas report on this system, design studies were carried out to determine the type of modifications of the heat exchangers and power generation equipment that might be required in order to interface the solar collector with a cavern storage system. Though some modifications would be necessary, none are considered to be of major significance.

Larger systems--100 MW<sub>e</sub> to 1000 MW<sub>e</sub>--would require a much more complex interfacing of the solar collector, cavern storage, and steam-electric turbines.

One proposed design for such larger systems is a cross-compound system using steam-electric conversion at two different temperature levels, one direct from the collector and one direct from the cavern, with cross coupling. This would be desirable since output from the solar tower would be at 900-1000°F while cavern storage is limited to about 650°F by the oil storage fluid.

Detailed cost estimates were prepared for the 10 MW<sub>e</sub> system and some preliminary estimates of costs for larger systems were made; these are shown below.

Economics. The total cost, C, of a cavern storage system is given by

\[ C = c_c V + C_w + C_D + C_a \]

where

\( c_c \) = unit volume cost of cavern and contents (gravel, oil)

\( V \) = cavern volume

\( C_w \) = costs of hot and cold wells

\( C_D \) = cost of brine disposal well

\( C_a \) = cost of above ground equipment.

For systems of the size envisioned in this study, one brine disposal well \((C_D = $620,000)\) would be adequate for the cavern leaching operation. The unit volume cost of the cavern and its contents is approximately $2.85/ft<sup>3</sup>. This includes costs of leaching, gravel placement, and oil. The cavern volume is dependent upon the desired storage capacity while \( C_w \) and \( C_a \) depend upon the desired transfer rate.
Cost estimates for the components of a cavern system with a transfer rate of 33 MW_t and 8-hour storage period, given in various sections of the complete second year report, are summarized below:

Total storage system costs for 33 MW_t transfer rate and 8-hour storage period. million $

<table>
<thead>
<tr>
<th>Component</th>
<th>Cost</th>
</tr>
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<tbody>
<tr>
<td>Cavern Contents (139,000 ft³ @ c_c = $2.85)</td>
<td>0.396</td>
</tr>
<tr>
<td>C_w (1 hot well, 1 cold well)</td>
<td>1.638</td>
</tr>
<tr>
<td>C_D (1 brine disposal well)</td>
<td>0.620</td>
</tr>
<tr>
<td>C_a (heat exchangers, pumps)</td>
<td>0.733</td>
</tr>
<tr>
<td><strong>TOTAL</strong></td>
<td><strong>3.387</strong></td>
</tr>
</tbody>
</table>

The total cost of this system is approximately $3.4 million. This sum corresponds to total storage system costs of $103/kW_t and $13/kWh_t. These figures compare favorably with Department of Energy cost goals for near term sensible heat storage. Cost goals (C_t) can be compared to power related costs (C_p) and capacity related costs (C_s). Power related cost (C_p) depends upon the capability of the storage system to accept and deliver thermal energy at a given rate (heat exchangers, wells, pumps, etc.). Capacity related cost (C_s) is related to the maximum amount of energy that can be contained within storage (oil, gravel, construction costs of cavern capacity). These relations are shown below:

<table>
<thead>
<tr>
<th>Storage System Costs</th>
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<tbody>
<tr>
<td>C_t ($)/kW_t</td>
</tr>
<tr>
<td>----------------</td>
</tr>
<tr>
<td>DOE Goals 90</td>
</tr>
<tr>
<td>DOE Goals* 105</td>
</tr>
<tr>
<td>This study 103 (small system)</td>
</tr>
</tbody>
</table>

Here: $C_t($)/kW_t = C_p + h·C_s$

$C_t($)$/kWh_t = (C_p + h·C_s)/h

*DOE 6 hour costs converted to 8 hour costs for direct comparison with this study.
The underground storage value for the power cost is about twice the DOE goal, but the capacity cost is much less. This difference exists because the underground system has very low containment costs which reduce capacity related costs ($C_s$), but the power related costs ($C_p$) are increased primarily due to the well costs.

Cost figures quoted are for a minimum underground system, costs for larger commercial scale systems would be less. Minor modifications in well design would conceivably allow doubling the flow rates used in this study. With this change, the cost of a large storage system (100 MW\textsubscript{e} or larger) would be approximately $60/kW_t$ or $7.50 \text{kWh}_t$. Therefore, cavern storage appears to be an attractive option for near term sensible heat storage for solar power systems of large size. Cavern storage may also be economically favorable for storage periods long enough (16 hours) to provide baseline electric power.