The object of the invention is to provide a low compression turbocharged diesel engine in which the turbocharger can be operated independently of the engine to power auxiliary equipment. As shown in FIG. 1, fuel and air are burned in a catalytic combustor 17 to drive the turbine wheel of turbine section 110 which is initially caused to rotate by starter motor 9. By opening flapper valve 26, compressed air from blower section 116 is directed to catalytic combustor 17 when it is heated and expanded, serving to drive the turbine wheel and also to heat the catalytic element 32 shown in FIG. 2. To start engine 10, valve 26 is closed, combustion is terminated in catalytic combustor 17, and valve 20 is then opened to utilize air from blower section 11b for air driven motor 21. When engine 10 starts, the constituents in its exhaust gas react in the catalytic element 32 and the heat generated provides additional energy for the turbine section 11a.

3 Claims, 2 Drawing Figures
4,449,370

DIESEL ENGINE CATALYTIC COMBUSTOR SYSTEM

DESCRIPTION

Origin of the Invention

The invention described herein was made by employees of the United States Government and may be manufactured and used by or for the Government for governmental purposes without the payment of any royalties thereon or therefor.

TECHNICAL FIELD

This invention relates to turbocharged internal combustion engines and is directed more particularly to a turbocharged diesel engine for aircraft use.

Because of the urgent need to reduce the amount of crude oil imported to the United States, a high priority has been placed on increasing the fuel economy of various internal combustion engines including those used in light aircraft. It is well-known that the brake specific fuel consumption for diesel engines is much lower than for spark ignition type gasoline engines.

For aircraft use, diesel engines are considered to be safer than spark ignition engines because no electrical ignition system is used and, consequently there are no misfires. Further, a diesel can be turbocharged to operate as efficiently at a 24,000 ft. altitude as it would at ground level, an advantage which cannot be achieved either by turbine engines or by supercharged gasoline engines which are generally limited to about three pounds per square inch supercharging pressure. Where a turbocharger is used, hot air can be bled through a diesel engine to prevent cooldown as would occur in a descent mode.

Probable main disadvantage of a diesel engine as far as used in aircraft is its great weight which results from making the engine structure strong enough to withstand the pressures generated by compression ratios of 20 to 1 or more. However, by reducing the compression ratio to about 14 to 1 or as low as 8 to 1, the engine can be made very light. With such a low compression ratio, a turbocharger may be used to raise the intake air temperature to the 850°F needed to provide auto-ignition in the diesel engine. However, for starting purposes some means must be provided to activate the turbocharger before the engine is running.

An important aspect of light aircraft usage is the availability of electrical power from the aircraft itself while on the ground with the engine off. A turbocharger which can be operated without the engine running can be used to drive an alternator or generator to provide the needed electrical power. The turbocharger can also be used to drive an oil pump for circulating oil through the diesel engine before attempting to start it.

BACKGROUND ART

In the prior art U.S. Pat. No. 4,018,055 discloses an internal combustion engine with an exhaust turbocharger and having an external auxiliary combustion chamber which provides hot gas to the exhaust system of the engine. The hot gas from the auxiliary combustion chamber is directed toward the turbine wheel thereby increasing its speed so as to provide increased starting-supercharged air pressure as well as additional pressure required under acceleration or heavy load.

U.S. Pat. No. 3,775,971 discloses a turbocharged internal combustion engine having an after-burner disposed in the exhaust system between the engine and the turbine of a turbocharger. Fuel may be injected into the after-burner to increase the energy of the turbine wheel. Valves are provided to supply additional amounts of air to the after-burner and to recirculate some of the exhaust gas to the intake system of the engine.

Neither of the above-identified patents involves any use of a catalytic combustion with a turbocharger to drive auxiliary equipment when an aircraft is parked on the ground with the engine off and the arrangements of both patents involve problems of flame-out and restarting with regard to the after-burner and auxiliary combustion chamber.

DISCLOSURE OF INVENTION

In accordance with the invention turbocharged, lightweight diesel engine is provided with a starter/alternator and an oil pump mechanically driven from a shaft through which the turbine wheel of a turbocharger drives the turbocharger blower and has a catalytic combustor disposed in the exhaust system between the engine and the turbine wheel. A by-pass conduit extends from the exhaust system at a point upstream of the catalytic combustor to an intake conduit extending between the blower of the turbocharger and the intake of the engine and is provided with a flapper valve which can be positioned either to block the by-pass conduit and open the intake conduit or vice-versa.

By positioning the by-pass conduit valve to open the by-pass conduit but block the intake conduit and by burning fuel in the catalytic combustor to heat the catalytic element, energy is supplied to the turbine wheel, thereby rotating the blower and driving the alternator and the oil pump. Air from the blower wheel is diverted through the by-pass conduit to the exhaust system and the catalytic combustor and to the turbine wheel to the atmosphere.

After the catalytic element of the catalytic combustor achieves an operating temperature of approximately 400°F or greater, the fuel input to the combustor may be varied as needed and even reduced to zero, the combustor will provide automatic restart without any ignition device such as a spark. Also, during operation of the engine there is sufficient oxygen in the exhaust gas so that no air need be supplied to the combustor from an external source to support combustion.

BRIEF DESCRIPTION OF THE DRAWINGS

The details of the invention will be described in connection with the accompanying drawings in which FIG. 1 is a schematic drawing of a turbocharged engine embodying the invention, and FIG. 2 is a schematic of a catalytic combustor of the type used in the arrangement of FIG. 1.

BEST MODE FOR CARRYING OUT THE INVENTION

Referring first to FIG. 1, there is shown an intermittent combustion engine 10 having a turbocharger comprised of a turbine wheel section 14a, a blower section 11b, and a means for connecting the turbine to drive the blower such as shaft 11c. Air taken in at an air intake 12 is compressed and raised in temperature by the blower wheel section 11b and is directed by an intake conduit 13 to an intake port 14 of the engine 10. Exhaust gases from the engine 10 exit at exhaust port 15 and pass through an exhaust conduit 16, a catalytic combustor
17, the turbine section 11a and then through an exhaust
outlet 18 to atmosphere.

The temperature of the intake air may be kept within
a desired range by a intercooler 19 disposed in the in-
take conduit. Also, compressed air may be tapped off
from the intake conduit 13 through a valve 20 to a
pneumatic starter motor 21 to provide cranking power
for engine 10. Further, by running the turbocarhger
while engine 10 is inoperative, accessory equipment
such as an oil pump 22 and a generator 23 may be driven
from the shaft 11c by means of various known mechan-
ical devices as, for example, a gear box 24. With this
arrangement it is necessary to provide a means such as
a starter motor 9 to rotate the turbine wheel and blower
initially. Alternatively, starter 9, gear box 24, oil pump
22 and generator 23 may be replaced by an alternator-oil
pump in shaft 11c. The alternator serves as a starter
motor when electrical power is supplied thereto.

To the end that turbine section 11a will drive accesso-
ries 22 and 23 and the blower 11b while engine 10 is
inoperative, a bypass conduit 25 is connected from the
exhaust conduit 16 upstream of catalytic combustor 17
to the intake conduit 13. In order to block the intake
conduit 13 and direct air from the blower 11b through
the by-pass conduit 25 to the combustor when the tur-
bocharger is to be operated while engine 10 is inactive,
a flapper valve 26 is positioned at the juncture of by-
pass conduit 25 and intake conduit 13.

When engine 10 is inoperative, hot gas is supplied
from the catalytic combustor 17 to the turbine section
11a by burning fuel in the combustor 17. This fuel is
supplied from a fuel source 27 through a valve 28. Air
to support the combustion is supplied from an air source
29 through a valve 30 to the catalytic combustor 17.

Referring now to FIG. 2 there is shown in schematic
detail a catalytic combustor in accordance with the
invention. Catalytic combustor 17 comprises a hous-
ing 31 closed at one end and having a catalytic element 32
disposed in an open end, the open end being connected
through exhaust conduit 16 to the turbocarhger turbine
section 11a. The catalytic element 32 may be of palla-
dium or other noble metals capable of reacting the con-
tituents of exhaust gas from an internal combustion
engine. Apertures or slots 33 are provided in the hous-
ing 31 and are enclosed within an annular manifold 34
which receives exhaust gas from exhaust conduit 16 or
air from by-pass conduit 25.

Between the apertures 33 and the closed end of hous-
ing 31 there is disposed a combustion chamber 35 into
which fuel is injected by a fuel nozzle 36. Air is supplied
to combustion chamber 33 from valve 30. The fuel
spray from injector 36 is ignited in combustion chamber
35 by a spark gap 36 energized from an igniter power
source 37 to produce a flame 38. A flameholder 39 is
disposed coaxially with the combustion chamber 35 to
inhibit flameout conditions and to distribute the hot gas
generated by the flame uniformly to the catalytic ele-
ment 32.

Operation of the engine 10 and turbocarhger system
of FIG. 1 will now be described. Assuming that engine
10 is inoperative, the turbocarhger section 11a may be
energized activating starter motor 9 and by providing
fuel and air to the catalytic combustor 17 to produce hot
gas. Rotation of the turbine wheel of turbine section 11a
drives not only the blower wheel of blower section 11b
but also the oil pump 22 and the generator 23.

With flapper valve 26 positioned to block intake con-
duit 13, compressed air from the blower 11b is directed
through by-pass conduit 25 to the catalytic combustor
17 where it is further raised in temperature and heats the
catalytic element 32 before passing through turbine
wheel section 11a and out through exhaust outlet 18.

After the catalytic element 32 is brought up to about
400° F., the fuel and air to the catalytic combustor 17
are adjusted by means of valves 28 and 30 to provide the
desired thermal output from the combustor.

After catalytic element 32 is up to the desired tem-
perature, flapper valve 26 is positioned to close the by-pass
conduit 25 and open intake conduit 13. Compressed air
from the blower section 11b may be bled off from intake
conduit 13 to a pneumatic motor 21 by opening valve
20. The compressed air to motor 21 causes it to crank
the engine 10 until the latter starts.

After engine 10 starts, valve 30 may be closed, shut-
ting off air to the catalytic combustor 17 since the ex-
haust gas normally contains sufficient oxygen to sup-
port combustion in the catalytic combustor. Eventually
valve 28 may also be closed because the catalytic ele-
ment 32 adds additional heat to the exhaust gas and,
consequently, provides additional power to drive the
wheel of turbine section 11a.

Whenever needed, fuel and air can again be provided
to the catalytic combustor to provide additional heating
of the exhaust gas. To avoid manual control of valves 28
and 30, a control module 40 may be provided as shown in
FIG. 1. Input signals are directed to control module
40 from a temperature sensor 41 in catalytic combustor
17 and from an engine running sensor 42 on engine 10
via leads 43 and 44, respectively. Sensor 42 may be, for
example, a switch which energizes module 40 from a
battery or it may take the form of a generator which
provides electrical power to module 40 when engine 10
is operating. Control signals are supplied from control
module 40 to valves 28 and 30 via leads 45 and 46,
respectively. The temperature sensor 41 signal causes
control module 40 to gradually close valves 28 and 30 as
the catalytic element 32 (FIG. 2) approaches its normal
operating temperature. When engine 10 is started, sen-
ator 42 signals control module 40, whereby if the cata-
litic element of catalytic converter 17 is up to its nor-
amal temperature, both valves 28 and 30 will close. The
catalytic element is kept hot by the reactive constituents
in the exhaust gas from engine 10.

It will be understood that those skillled in the art to
which the invention pertains may make various changes
and modifications to the above described invention
without departing from the spirit and scope thereof as
set forth in the claims appended hereto.

I claim:

1. In combination with an intermittent combustible
engine of the type having a turbocarhger with a turbine
wheel driven by engine exhaust gases directed thereto
by an exhaust conduit and with a blower which pro-
vides air under pressure to said engine through an intake
conduit, said blower being mechanically connected to
said turbine:

(1) a catalytic combustor disposed in said exhaust
conduit, and comprising:

- an elongated housing having a closed end and an
  open end, said open end communicating with
  said turbine wheel, and further having slots
  which communicate with said engine through
  said exhaust conduit, said slots being disposed in
  said housing in the form of an annular row;
- a catalytic element disposed at the open end of said
catalytic combustor;
a cylindrical combustion chamber disposed between the closed end of said elongated housing and said slots which communicate with said engine;
means for injecting fuel into said combustion chamber;
means for injecting air into said combustion chamber;
means for igniting said fuel injected into said combustion chamber;
a flared apertured tube disposed coaxially outwardly of said combustion chamber and extending toward said slots which communicate with said engine to inhibit flameout and to uniformly heat said catalytic element;
(2) a fuel supply connected to said means for injecting fuel through a first valve which may be closed to terminate injection of fuel into said combustion chamber;
(3) an air supply connected to said means for injecting air through a second valve which may be closed to terminate injection of air into said combustion chamber;
(4) means for automatically adjusting said first and second valves when said catalytic element attains a predetermined temperature whereby overheating of said catalytic element is prevented;
(5) means coacting with said means for adjusting said first and second valves to close said first and second valves when said catalytic element is at or above a temperature of 400°F. and said engine is operating whereby combustor restart is automatic when said first and second valves reopen; and,
(6) an intercooler disposed in said intake conduit to reduce the temperature of the air being provided to said engine.
2. The engine-turbocharger system of claim 1 wherein said catalytic element is selected from the group of noble metals which will minimize hydrocarbon emissions from the exhaust conduit by catalyzing a chemical reaction between the constituents of the exhaust gas.
3. The engine-turbocharger system of claim 2 wherein said catalytic element is palladium.