A method and a system for testing the bonding of foam insulation (22) attached to metal. The system involves the use of an impacter (10) which has a calibrated load cell (12) mounted on a plunger (14) and a hammer head (16) mounted on the end of the plunger (14). When the impacter (10) strikes the insulation (22) at a point to be tested, the load cell (12) measures the force of the impact and the precise time interval during which the hammer head (16) is in contact with the insulation (22). This information is transmitted as an electrical signal (20) to a load cell amplifier (28) where the signal (20) is conditioned and then transmitted to a Fast Fourier Transform (FFT) analyzer (34). The FFT analyzer (34) produces energy spectral density curves (power plotted against frequency in Hertz) which are displayed on a video screen (38). An operator, by observing the frequency point at which the curve terminates, may determine the quality of the bond. Specifically, the termination frequency of the energy spectral density curve may be compared with a predetermined empirical scale to determine whether a high quality bond, good bond, or debond is present at the point of impact. For future reference and use, data from the FFT analyzer (34) are recorded on a magnetic disk (41) and/or a hard copy is produced by a printer (43).
ABOUT .008
AVERAGE TIME, SEC. FIG. 3

FORCED PERIOD OF VIBRATION

ENERGY SPECTRAL DENSITY
WELL BONDED

FREQUENCY, HZ FIG. 4(a)

ENERGY SPECTRAL DENSITY BONDED

FREQUENCY, HZ FIG. 4(b)

ENERGY SPECTRAL DENSITY DEBONDED

FREQUENCY, HZ FIG. 4(c)
INSULATION BONDING TEST SYSTEM

ORIGIN OF THE INVENTION

The invention described herein was made in the performance of work under a NASA contract and is subject to the provisions of Section 305 (72 Stat. 435; 42 USC 2457).

BACKGROUND OF THE INVENTION

The present invention relates generally to non-destructive testing, and more particularly to a method and apparatus for testing the bonding of foam insulation attached to metal. Specifically, the invention is for testing the bonding of foam insulation sprayed on metal cryogenic fuel tanks.

CROSS REFERENCE TO RELATED APPLICATION

This application is related to U.S. patent application Ser. No. 460,509 filed Jan. 24, 1983, entitled "Impacting Device For Testing Insulation."

TECHNICAL FIELD

The present invention is a method and apparatus for testing the bonding of foam insulation sprayed on metal cryogenic fuel tanks. The present invention results from efforts made by the assignee of the present application in the field of impact testing of insulating materials, and particularly in the field of testing the bonding of foam insulation attached to metal. The present invention is also related to the prior art patents which disclose a device which could accomplish the required prior art patents have at least some utility as non-destructive test devices. However, none of the prior art patents disclose a device which could accomplish the purpose which is required of the instant invention. These prior art devices all appear to use a resonant or variable frequency oscillator and an electro-mechanical non-destructive insulation bond test system which would operate on one side of the workpiece only to impact to the workpiece is actually taking place. Moreover, the present invention does not use a micro-phone to sense resonant sound waves as does Batterman and Pezzillo.

SUMMARY OF THE INVENTION

The present invention is a method and apparatus for non-destructive testing of foam insulation attached to metal. The apparatus comprises a lightweight hammer.
for impacting insulation attached to metal, force measuring means mounted on the hammer for measuring the force with which the hammer strikes the insulation, means connected to the force measuring means for determining the amplitude and forced period of vibration of mechanical waves set up when the hammer strikes the insulation, and means connected to the determining means for showing a visual presentation of an energy spectral density curve for said mechanical waves. A comparison may be made between the energy spectral density curve and a predetermined empirical scale and a determination made as to whether the insulation at the point of impact is well bonded, bonded, or debonded. The method comprises the steps of: impacting insulation on metal with a calibrated impacter measuring the force of the impact and the time of the resulting force vibration, electronically generating an energy spectral density curve and visually presenting said curve on its screen, determining the frequency at which the energy spectral density curve terminates, and comparing the termination frequency of the energy spectral density curve with a predetermined empirical scale to determine whether a high quality bond, good bond, or debond is present at the point of impact.

BRIEF DESCRIPTION OF THE DRAWINGS

A presently preferred embodiment of the invention will now be described in detail in connection with the accompanying drawings, wherein:

FIG. 1 is a schematic diagram of a part of the invention in position to impact insulation in order to obtain basic data.

FIG. 2 is a schematic diagram of the invention.

FIG. 3 is a graph showing a history of load cell measurement.

FIG. 4a is a graph showing the energy spectral density curve of well bonded insulation.

FIG. 4b is a graph showing the energy spectral density curve of bonded insulation.

FIG. 4c is a graph showing the energy spectral density curve of debonded insulation.

FIG. 5 is an assembly drawing of the invention shown partially in section and with the housing partially cut away in order to show the parts.

FIG. 6 is an enlarged cut-away view showing how the sear holds the plunger in its cocked position.

DETAILED DESCRIPTION OF THE INVENTION

FIG. 1 is a schematic diagram of part of a preferred embodiment of the present invention. The basic equipment used in the prototype system to obtain the data needed to evaluate the bond between insulation and a metal cryogenic fuel tank is shown. The system involves the use of an impacter or lightweight hammer, shown generally at 10, which has a calibrated load cell 12 mounted on a moving rod or plunger 14. The load cell used in the prototype device is PCB Model 208A03, which is commercially available from Piezotronics, Inc. The power supply for the load cell is PCB Model 480D06. An impact hammer head 16 is mounted at the end of plunger 14. Cable 18 carries force measurement data 20 from load cell 12 to other elements of the system for processing and use in evaluating the quality of bonding of the insulation as will be described hereinafter.

As can be seen in FIG. 1, lightweight hammer 10 is poised, ready to impact a layer of spray-on foam insulation 22. Insulation 22 is attached to a layer of an ablating material 24, which in turn is attached to a metal substrate 26. This metal substrate in this instance is the wall of an aluminum alloy fuel tank (the external tank of the Space Shuttle). The ablating material is applied under the foam to the front and back ends of the external tank and to other areas where aerodynamic heating is most severe during flight. The ablating material is designed to prevent heat transfer to the cryogenic fuel in the tanks.

Looking now at FIG. 2, there is shown a block diagram of the bond test system. In addition to the parts of the system already discussed for FIG. 1, it will be noted that cable 18 carries force measurement data 20 to a load cell amplifier 28. Amplifier 28 conditions data 20 and changes it to a voltage signal which is compatible with analyzer 34. The output 30 of load cell amplifier 28 is fed to a Fast Fourier Transform (FFT) analyzer 34. FFT analyzer 34 is connected by a cable 36 to a desktop computer 38 which has a video display 39 and mass storage compartment 41. FFT analyzer 34 is a commercially available Hewlett-Packard structural dynamics analyzer, Model HP5423A, connected to a Hewlett-Packard Model HP9826 desktop computer 38. A Hewlett-Packard Model HP2673A printer 43 is connected to computer 38. As indicated above, one of the basic concepts of this invention involves the use of an impacting device 10 to apply a very low force impact to the insulation. Load cell 12 measures the force and duration of the impact and transmits data 20 to remote equipment where it may be read out as a single pulse. The period of time during which the impact is being applied to the insulation is called the forced vibration period. For the type of sprayed-on foam insulation presently being used on the external tank of the Space Shuttle, this period is 20.15 seconds in the range of 5 to 12 milliseconds in duration and averages about 8 milliseconds. It should be noted that this forced vibration period will vary with different insulation materials, thickness of the insulation, different bonding materials, and so forth. Load cell 12 senses only during this forced vibration period that, after the force is removed, there is no further signal on the load cell. The signal generated by load cell 12 is a direct and instantaneous measure of the local stiffness of insulating material 22 at the point of impact. If insulation 22 is well bonded to the metal substrate, the stiffness or impedance will be much higher, causing a shorter duration pulse. The energy spectral density curve (or power spectrum curve) of the pulse obtained from a bonded point will display frequencies of 440 Hertz and above. The energy spectral density curve of the pulse obtained from a very well bonded point will display frequencies of 520 Hertz and above. The energy spectral density curve of the pulse obtained from a debonded point will display only frequencies below 360 Hertz because the insulation alone (not bonded) does not have the stiffness to support energy at the high frequencies. This will be explained in more detail with regard to FIGS. 3 and 4 below.

FIG. 3 shows a typical time history of the forced period of vibration as it is fed to the on-line Fast Fourier Transform (FFT) analyzer 34. Fast Fourier Transform analyzer 34 takes this time history and rapidly converts it to its respective amplitude versus frequency curve in order to enable the operator to make the final decision as to the condition of the bonding. This frequency conversion made by the Fast Fourier Transform analyzer takes less than one second.
FIGS. 4c, 4b, and 4c show typical cases of energy spectral density curves for areas which are well bonded, bonded, and debonded, respectively. These curves produced by the FFT analyzer are arranged to plot a unit of power (pounds-seconds/Hertz or db) against frequency in Hertz. When the frequency conversion is made by the FFT analyzer, the operator need only observe one feature of the curve to determine the quality of the bond. First, he observes the point at which the curve terminates. As may be seen in FIG. 4c, if it is above 520 Hertz for thick insulation, this is an indication of excellent bonding. If the curve terminates in the range from 440 Hertz to 520 Hertz, this is an indication of good bonding, as shown in FIG. 4b. Conversely, when the energy spectral density curve terminates below 360 Hertz, it is an indication that a debond exists, as shown in FIG. 4c.

FIG. 5 shows an assembly drawing of the preferred embodiment of impactor 10, with part of the device in section, and with part of case 27 cut away in order to show the arrangement of the parts. Case 27 is designed to be approximately the same size and shape as the U.S. Army .45-caliber automatic pistol. Plunger 14 is a one-piece, long metal rod which extends completely through the upper part of case 27 and protrudes from both the front and back ends of case 27. Plunger 14 may be cocked by grasping the two guides 29 located at the rear end of plunger 14 and pulling plunger 14 to the rear, thus compressing the front or driving spring 31. When plunger 14 is released by trigger 33, driving spring 31 drives the plunger forward so that it may perform its impacting function.

Plunger 14 is supported in case 27 by a forward bushing 35 mounted in end fitting 37 and by rear bushing 40 located directly above the handle. Load cell 12 is attached to the front end of plunger 14. Hammer head 16, which preferably is made of an epoxy fiberglass, may be formed with a flat front impacting surface, as shown in FIG. 5. Preferably, however, the front impacting surface of hammer head 16 is slightly rounded so that the lightweight hammer will deliver uniform impacting blows even if the impactor is canted at a slight angle to the workpiece.

The front or driving spring 31 is located in a circular 45 driving spring reces 42 located in the rear end of the barrel area of impactor 10. Driving spring 31 is retained under compression by front spring stop 44 and by the rear surface 46 of the barrel area. Front spring 31 may be provided in several wire diameters, giving several different spring rates, as desired.

Rear spring 48 surrounds plunger 14 in the upper rear section of case 27. Rear spring 48 provides a cushioning effect when front spring 31 drives the plunger forward. As plunger 14 travels forward under the driving force of front spring 31, rear spring 48 is compressed, thereby slowing the plunger down immediately after hammer head 16 impacts the workpiece. This prevents hammer head 16 from bouncing or impacting insulation 22 more than once. Spring 48 is retained under compression by front bushing 40, which also acts as a forward stop for spring 48 and by stop 50, which is attached to the rear end of plunger 14. As plunger 14 goes forward, stop 50 contacts spring 48 and compresses it, thus providing the aforementioned cushioning effect.

Impactor 10 has a precisely adjustable trigger mechanism. When trigger 33 is pulled to the rear, trigger projection 52 contacts trigger spring 54 and compresses it. At the same time, trigger projection 52 takes the lower end of trigger link 56 to the rear with it. Trigger link 56 is pivotally connected by upper trigger link pin 60 to rear 62. A pull on trigger link 56 exerts a downward pull on rear 62 and causes the tip of rear 62 to come out of machined notch 65 at the rear end of machined step 64 in plunger 14.

The trigger mechanism has a very precise adjustment. When set screw 66 is turned outward, the forward end of sear adjuster 68 is caused to move upward, and sear adjuster 68 rotates about pin 70. This causes the rear end of sear adjuster 68 to push downward on rear 62, thus adjusting the precise point where the tip of sear 62 contacts the rear end of machined step 64 in plunger 14. Preferably, the tip of rear 62 will be seated in notch 65. This provides a very precise adjustment on both trigger pull and operation of the trigger mechanism.

To provide stability to impactor 10 during the time when the device is impacting the insulation, location guide assembly 72 is attached to the front end of the barrel area. Sleeve 74 fits over the outside of the barrel and is fastened to the barrel by two set screws 76. Extending straight out from circular sleeve 74 are two location guide rods 78. These location guide rods 78 extend out from circular sleeve 74 about three to four inches on the prototype device and are then bent forward at a 90° angle. Bumper 80 is attached to bumper rod 82, which may be slidably attached to each of the two location guide rods 78. Guide rod sleeve 84 holds each of the two bumper rods 82 on its respective guide rod 78. Each guide rod sleeve 84 is attached to its respective guide rod 78 by a pin 88 attached to guide rod 78. Each pin 88 rides on a slot 90 in each guide rod sleeve 84. When impactor 10 is pushed against the workpiece, bumpers 80 contact the workpiece first and cause bumper rods 82 to compress guide rod springs 86. As will be apparent from the above discussion of the design of the location guide assembly, sleeve 74 may be oriented so that location guide rods 78 are either at the sides of the barrel area or above and below it, whichever may be desired by the operator.

An additional gauge on impactor 10 is provided by gauge rod 92 which is mounted at the end of the barrel area. Rod 92 is attached to end fitting 38 by bracket 94 and studs 96. Rod 92 may be slid forward and backward by adjusting set screw 98. This provides a way for the operator to see and have a "feel" for the distance to hold impactor 10 from the insulation, as may be desired by the operator or required by the type of insulation being tested.

FIG. 6 is an enlarged cut-away view of the relationship between sear 62 and plunger 14 and shows how the sear holds the plunger in its cocked position.

Going now to the operation of the preferred embodiment of the system as shown in FIG. 2, impact hammer 10 is released from its cocked position by pulling trigger 34 so that hammer head 16 impacts sprayed-on foam insulation 22. For the duration of the time hammer head 16 is in contact with insulation 22 (forced period of vibration), load cell 12 senses data 20 and transmits it to amplifier 28. To avoid any possible loss of the data signal, the FFT analyzer is set up in a pre-trigger mode. A conditioned signal is then transmitted to FFT analyzer 34, which is connected to a desktop computer 38 with video display on screen 39. The frequency at which the energy spectral density curve terminates is compared with an empirical scale to enable the operator to make a determination of whether the point of impact
is well bonded, bonded, or debonded. Data from FFT analyzer 34 may also be recorded in permanent form on a disk. Optionally, this data may also be dumped to printer 43 to produce a hard copy.

In practice, the system is used to test a large number of points in succession, using a matrix and following a set procedure. Impact points in the matrix are tested in a particular pre-arranged order and following a particular pattern. Computer 38 keeps a record of the test results for each impact point on the insulation according to its coordinates in the matrix. When the testing is complete, computer 38 can map the tested area and indicate bonded and debonded areas.

The present invention has been specifically described for purposes of illustration as being used in testing portions of the exterior walls of the external tank of the Space Shuttle, consisting of curved metal plates having a foam composite sprayed on them. However, it is obvious that the invention also finds utility in the testing of panels composed of a wide variety of other kinds of materials.

The nature and scope of the present invention having been indicated and the preferred embodiment of the invention having been specifically described, what is claimed is:

1. A system for testing the bonding of insulation attached to metal comprising:
a hammer for impacting insulation attached to metal, said hammer comprising means for measuring the force with which said hammer strikes said insulation and the time duration of the impact and generating an electrical signal representing said force and said time duration;
conditioning means for transforming said electrical signal by amplifying the voltage, said conditioning means having a high input impedance and a low output impedance; and
means connected to said conditioning means for using said conditioned signal for determining spectral data representing the amplitude and forced period of vibration of mechanical waves set up when said hammer strikes said insulation and transforming said data into data representing an energy spectral density curve for said mechanical waves, said determining means comprising means for showing a visual presentation of an energy spectral density curve;
whereby, a comparison may be made between said energy spectral density curve and a predetermined empirical scale and a determination made as to whether: insulation at the point of impact is well bonded, bonded, or debonded.

2. The system of claim 1 wherein said hammer comprises a spring-loaded plunger.

3. The system of claim 2 wherein said measuring means comprises a load cell mounted on said plunger.

4. The system of claim 1 wherein said conditioning means comprises a load cell amplifier.

5. The system of claim 4 wherein said determining means is a Fast Fourier Transform analyzer.

6. The system of claim 5 comprising recording means for recording data from said conditioning means.

7. The system of claim 6 wherein said measuring means comprises a magnetic disk for recording data from said determining means.

8. The system of claim 7 wherein said hammer comprises a load cell mounted on said plunger.

9. The system of claim 8 wherein said recording means comprises a printer for recording data from said determining means.

10. The system of claim 9 wherein said determining means comprises a printer for recording data from said determining means.

11. A method for testing the bonding of insulation attached to metal comprising the steps of:
imparting insulation attached to metal with a calibrated impacter measuring the force of the impact and the time duration of the resulting force vibration;
generating an electrical signal which is representative of said force and said time duration;
electronically transforming said electrical signal into an energy spectral density curve and visually presenting said curve on a screen;
determining the frequency at which said energy spectral density curve terminates; and
comparing the termination frequency of said energy spectral density curve with a predetermined empirical scale to determine whether a high quality bond, bond, or debond is present at the point of impact.

12. The method of claim 11 comprising conditioning said electrical signal before electronically transforming said electrical signal to said energy spectral density curve.

13. The method of claim 12 comprising electromagnetically recording the data from said energy spectral density curve in order to preserve it.

14. The method of claim 12 including using a printer for recording the data from said energy spectral density curve in order to preserve it.