An Exponential Finite Difference Technique for Solving Partial Differential Equations

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PARTIAL DIFFERENTIAL EQUATIONS

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ABSTRACT

An exponential finite difference algorithm, as first presented by Bhattacharya for one-dimensional unsteady state, heat conduction in Cartesian coordinates, has been extended. The finite difference algorithm developed was used to solve the diffusion equation in one-dimensional cylindrical coordinates and applied to two- and three-dimensional problems in Cartesian coordinates. The method was also used to solve nonlinear partial differential equations in one (Burger's equation) and two (Boundary Layer equations) dimensional Cartesian coordinates. Predicted results were compared to exact solutions where available, or to results obtained by other numerical methods. It was found that the exponential finite difference method produced results that were more accurate than those obtained by other numerical methods, especially during the initial transient portion of the solution. Other applications made using the exponential finite difference technique included unsteady one-dimensional heat transfer with temperature varying thermal conductivity and the development of the temperature field in a laminar Couette flow.
ACKNOWLEDGEMENTS

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NOMENCLATURE

\( a_1, b_1, c_1 \) Thomas algorithm variables

\( B \) Biot number

\( C_p \) material specific heat, J/kg \( \cdot \) K (Btu/lb \( \cdot \) °F)

\( h \) convection heat transfer coefficient, W/M\(^2\) \( \cdot \) °C (Btu/ft\(^2\) \( \cdot \) hr \( \cdot \) °F)

\( i,j,k \) nodal location in x, y, and z spatial coordinate directions respectively

\( J_0, J_1 \) Bessel functions of zero and first order

\( k \) thermal conductivity, W/M \( \cdot \) °C (Btu/hr \( \cdot \) °F \( \cdot \) ft)

\( k_i^q \) thermal conductivity at \( i^{th} \) position, \( n^{th} \) time step, W/M \( \cdot \) °C (Btu/hr \( \cdot \) °F \( \cdot \) ft)

\( L \) distance between plates, M (ft)

\( M \) dimensionless drive number

\( m \) number of sub-intervals

\( N \) number of nodes in a spatial direction

\( n \) time step position designation

\( q \) heat flux, W/M\(^2\) (Btu/hr \( \cdot \) ft\(^2\))

\( r \) spatial coordinate; cylindrical coordinates, M (ft)

\( T \) temperature, °C (°F)

\( t \) time, sec

\( \Delta t \) time between time steps \( n \) and \( n+1 \)

\( U \) Couette flow velocity, M/s (ft/s)

\( x,y,z \) spatial coordinates, Cartesian coordinates, M (ft)

\( \Delta x, \Delta y, \Delta z \) distance between nodal positions in the x, y, and z spatial directions

\( \alpha \) thermal diffusivity, M\(^2\)/s (ft\(^2\)/s)

\( \beta \) rate of thermal conductivity variation

\( \gamma \) \( \frac{\Delta t}{\rho C_p (\Delta x)^2} \); \((W/M \cdot \circ C)^{-1}\) \(((Btu/hr \cdot \circ F \cdot ft^2)^{-1})\)
<table>
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<td>( \kappa )</td>
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I. INTRODUCTION

Partial differential equations have many important applications in the fields of engineering and physics. Many exact solutions exist depending on, the partial differential equation, the boundary conditions, and the number of spatial dimensions under consideration [1]. Coordinate systems other than Cartesian, more than one spatial dimension, and the boundary conditions all can pose problems that are either extremely difficult or impossible, to solve by analytical methods. Numerical methods thusly become the only possible solution method if the problem complexity is not to be compromised. Typically the ability of a particular method to predict a field variable is tested by numerically solving a problem for which a known exact solution is available. The ability of the method to predict the exact results is a measure of the confidence that can placed in a solution where no exact solution exists or experimental test results are unavailable.

The objective of the work to be presented is to extend, expand, and compare an explicit exponential finite difference technique first proposed by Bhattacharya [2]. To date the method has only been used for one-dimensional unsteady-state, heat transfer problems in Cartesian coordinates.

The method has been expanded, in this report, to allow application to a variety of problems. The exponential method will be extended here to the case of one-dimensional unsteady heat transfer in cylindrical
coordinates. Also, it was used in two- and three-dimensional unsteady heat condition. Other cases of interest that were solved using this approach include temperature varying conductivity in one-dimensional heat transfer and the development of the temperature field in laminar Couette flow. Solutions of the above cases were either compared to exact solutions or to results obtained by alternative numerical techniques.

One final application of the exponential finite difference algorithm was made for nonlinear partial differential equations. Burger's equation along with the boundary layer equations are solved using the exponential method. Thus, a demonstration of how to apply the method to nonlinear problems is described.

The results of all the different cases considered in this study indicated that the exponential technique produced results that were more accurate than those found through other numerical techniques. All exponential computer codes and those of the other competing numerical analysis were run in double precision on either the IBM-3033 or the Cray X-MP mainframes. The computer codes developed for the exponential finite difference method and other numerical techniques used for comparison are contained in the appendix of this report.
II. - ANALYSIS

The Exponential Finite Difference Algorithm

An explicit exponential finite difference algorithm as first derived by Bhattacharya [2] will now be presented. The method can be applied to many of the partial differential equations found in engineering and physics. The diffusion equation as it applies to conduction heat transfer will be used in the demonstration that follows. In reference [2-3] the method was derived for one-dimensional conduction heat transfer in Cartesian coordinates. To show how the method can be extended, a derivation parallel to the one presented in reference [2] will be made for unsteady state heat conduction in two-dimensions. Equations of this type are typically solved numerically by a variety of methods [4].

For two-dimensional heat transfer in Cartesian coordinates with constant material properties, the appropriate partial differential equation is [5]:

\[ \frac{\partial T}{\partial t} = \alpha \left( \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} \right) \]  

To initiate the exponential method a product solution is assumed and is written as:

\[ T(x,y,t) = \phi(x)\psi(y)\theta(t) \]  

The initial conditions of the problem are assumed to be

\[ T(x,y,0) = f(x,y) \]
\[ \theta(0) = 1 \]
Now taking the appropriate deviatives of Eq. (2) with respect to the independent variables, yields
\[
\frac{\partial T}{\partial t} = \psi \phi \frac{\partial \phi}{\partial t} ; \quad \frac{\partial^2 T}{\partial x^2} = \psi \phi \frac{\partial^2 \phi}{\partial x^2} ; \quad \frac{\partial^2 T}{\partial y^2} = \phi \frac{\partial^2 \psi}{\partial y^2} \tag{4}
\]
Substituting Eq. (4) into Eq. (1) produces:
\[
\phi \psi \frac{\partial \phi}{\partial t} = \alpha \left\{ \psi \phi \frac{\partial^2 \phi}{\partial x^2} + \phi \psi \frac{\partial^2 \psi}{\partial y^2} \right\}
\]
Dividing both sides of the above by \( \phi \psi \) gives:
\[
\frac{1}{\phi \psi} \frac{\partial \phi}{\partial t} = \alpha \left\{ \frac{1}{\phi} \frac{\partial^2 \phi}{\partial x^2} + \frac{1}{\psi} \frac{\partial^2 \psi}{\partial y^2} \right\} = -\kappa \tag{5}
\]
It can be seen that the variables have been separated. Consequently both sides of Eq. (5) must equal a constant, say, \( \kappa \).

Now examining only the left hand side of Eq. (5),
\[
\frac{1}{\phi \psi} \frac{\partial \phi}{\partial t} = -\kappa
\]
Multiplying the left-hand side of this equation by \( \phi \psi/\phi \psi \) gives:
\[
\frac{\phi \psi}{\phi \psi} \frac{\partial \phi}{\partial t} = -\kappa
\]
which can be rewritten from Eq. (4) as:
\[
\frac{1}{T(x,y,t)} \frac{\partial T}{\partial t} = -\kappa
\]
Direct integration produces:
\[
T = c_2 \exp \left\{ -\kappa t \right\}
\]
Next, the initial condition is used to evaluate the integration constant giving \( c_2 = T(x,y,0) \); thus,
\[
T(x,y,t) = T(x,y,0) \exp \left\{ -\kappa t \right\} \tag{6}
\]
Returning to Eq. (5) only this time, the right-hand side is examined;
\[
\alpha \left\{ \frac{1}{\phi} \frac{\partial^2 \phi}{\partial x^2} + \frac{1}{\psi} \frac{\partial^2 \psi}{\partial y^2} \right\} = -\kappa
\]

Multiply this by \( \frac{\partial \psi}{\partial \phi} \) and obtain

\[
\alpha \left\{ \frac{\partial \psi}{\partial \phi} \frac{\partial^2 \phi}{\partial x^2} + \frac{\partial \phi}{\partial \psi} \frac{\partial^2 \psi}{\partial y^2} \right\} = -\kappa
\]

or

\[
\frac{\alpha}{\partial \phi} \left\{ \frac{\partial \psi}{\partial \phi} \frac{\partial^2 \phi}{\partial x^2} + \frac{\partial \phi}{\partial \psi} \frac{\partial^2 \psi}{\partial y^2} \right\} = -\kappa
\]

Equations (2) and (4) are now used which results in:

\[
\frac{\alpha}{\partial \phi} \left\{ \frac{\partial^2 T}{\partial x^2} + \frac{\partial T}{\partial y^2} \right\} = -\kappa
\]

The temperature appearing as a coefficient is replaced using its initial value so that,

\[
\frac{\alpha}{T(x,y,0)} \left\{ \frac{\partial^2 T}{\partial x^2} + \frac{\partial T}{\partial y^2} \right\} = -\kappa
\]

The partial derivative terms can be written in central difference form about a node \((i,j)\) as [6]:

\[
\begin{align*}
\frac{\partial^2 T}{\partial x^2} & \approx \frac{T^n_{i+1,j} + T^n_{i-1,j} - 2T^n_{i,j}}{(\Delta x)^2} \\
\frac{\partial^2 T}{\partial y^2} & \approx \frac{T^n_{i,j+1} + T^n_{i,j-1} - 2T^n_{i,j}}{(\Delta y)^2}
\end{align*}
\]

Thus Eq. (8) becomes:

\[
\frac{\alpha}{T^n_{i,j}} \left\{ \frac{T^n_{i+1,j} + T^n_{i-1,j} - 2T^n_{i,j}}{(\Delta x)^2} + \frac{T^n_{i,j+1} + T^n_{i,j-1} - 2T^n_{i,j}}{(\Delta y)^2} \right\} = -\kappa
\]
Now Eq. (10) is substituted to replace the constant $\kappa$ in the exponential of Eq. (6). Making the appropriate substitutions results in:

$$T_{i,j}^{n+1} = T_{i,j}^n \exp \left( \frac{\alpha \Delta t}{(\Delta x)^2} \left[ \frac{T_{i+1,j}^n + T_{i-1,j}^n + 2T_{i,j}^n}{(\Delta x)^2} \right] ight)$$

$$+ \left. \frac{T_{i,j+1}^n + T_{i,j-1}^n - 2T_{i,j}^n}{(\Delta y)^2} \right)$$

If the grid spacing is constant ($\Delta x = \Delta y$), then the above equation can be written as:

$$T_{i,j}^{n+1} = T_{i,j}^n \exp \left( \frac{\alpha \Delta t}{(\Delta x)^2} \left[ \frac{T_{i+1,j}^n + T_{i-1,j}^n + T_{i,j+1}^n + T_{i,j-1}^n - 4T_{i,j}^n}{(\Delta x)^2} \right] \right)$$

Note the $\Delta t$ that appears in the above is the time elapsed between time steps $n$ and $n+1$. The temperatures on the right-hand side are the four nearest neighbors to the $1,j$th node (see Fig. 1).

In keeping with the notation derived by Bhattacharya [2], the term

$$\Omega = \frac{\alpha \Delta t}{(\Delta x)^2}$$

is called the dimensionless time step and

$$M_{1,j}^n = \frac{T_{i+1,j}^n + T_{i-1,j}^n + T_{i,j+1}^n + T_{i,j-1}^n - 4T_{i,j}^n}{T_{i,j}^n}$$

is called the dimensionless drive number. Thus, Eq. (11) can be rewritten rather compactly as:

$$T_{i,j}^{n+1} = T_{i,j}^n \exp \left\{ \Omega M_{1,j}^n \right\}$$

It was found [2] that an improvement in the solution at the $n+1$ time step at node $(1,j)$ can be made if the time step is divided into
a number of equal length time sub-intervals. The method of time step sub-intervals can be described as follows. Let us assume that the time interval will be divided into three intervals including the one at the end of the time step (Fig. 2). Now returning to Eq. (14), and evaluating at the \( n+1/3 \) time step, results in:

\[
T_{1,j}^{n+1/3} = T_{1,j}^n \exp \left\{ \frac{\Omega}{3} M_{1,j}^n \right\}
\]

Proceeding from the \( n+1/3 \) time step to the \( n+2/3 \) time step:

\[
T_{1,j}^{n+2/3} = T_{1,j}^{n+1/3} \exp \left\{ \frac{\Omega}{3} M_{1,j}^{n+1/3} \right\}
\]

Finally, the temperature can be written at the \( n+1 \) time step:

\[
T_{1,j}^{n+1} = T_{1,j}^{n+2/3} \exp \left\{ \frac{\Omega}{3} M_{1,j}^{n+2/3} \right\}
\]

Now substituting Eq. (15) into Eq. (16) and substitution of Eq. (16) into Eq. (17), produces:

\[
T_{1,j}^{n+1} = T_{1,j}^n \exp \left\{ \frac{\Omega}{3} M_{1,j}^n \right\} \exp \left\{ \frac{\Omega}{3} M_{1,j}^{n+1/3} \right\} \exp \left\{ \frac{\Omega}{3} M_{1,j}^{n+2/3} \right\}
\]

or

\[
T_{1,j}^{n+1} = T_{1,j}^n \exp \left\{ \frac{\Omega}{3} \left[ M_{1,j}^n + M_{1,j}^{n+1/3} + M_{1,j}^{n+2/3} \right] \right\}
\]

where the \( M \)'s are then evaluated at the sub-time intervals and then summed for calculation of \( T \) at the \( n+1 \) time step. Equation (19) can be written more generally as:

\[
T_{1,j}^{n+1} = T_{1,j}^n \exp \left\{ \frac{\Omega}{m+1} \sum_{p=0}^{m} M_{1,j}^{n+p/(m+1)} \right\}
\]

In reference [2], it was shown that for heat transfer applications, the time step can be subdivided as follows:
The procedure necessary to determine the dimensionless drive numbers will now be described. Since the method is an explicit technique, all information is known from the previous time step or from the previous time sub-interval. The effort is then centered around calculation of the drive numbers at the requested number of time sub-intervals for each of the spatial positions (nodes). The calculation procedure is shown in Fig. (3). Because the drive numbers are evaluated at sub-time intervals, the temperature (or any other field variable) must also be known at these sub-time intervals. Therefore, the temperature field is calculated at each sub-time interval, and in turn is used to calculate the next drive number. The drive numbers for each node are summed for all the sub-time interval steps and then used to predict the field variable at the next complete time step. This results in a computer storage requirement of \(4(N)\), where \(N\) is the number of nodes.

**Extension To One-Dimensional Cylindrical Coordinates**

In the previous section, the exponential finite difference technique was extended to two-dimensions. Now the procedure will be considered in another coordinate system. In particular the method will next be applied to radial one-dimensional, unsteady heat transfer. The governing unsteady diffusion equation for a material with constant properties is:
\( \rho C_p \frac{\partial T}{\partial t} = k \left[ \frac{1}{r} \frac{\partial}{\partial r} \left( r \frac{\partial T}{\partial r} \right) \right] \)  

(22)

\[ \frac{\partial T}{\partial t} = \alpha \left[ \frac{\partial^2 T}{\partial r^2} + \frac{1}{r} \frac{\partial T}{\partial r} \right] \]

Assume that the temperature can be written as the product:

\[ T(r,t) = \phi(r) \theta(t) \]  

(23)

The initial conditions are

\[ T(r,0) = f(r) \quad \theta(0) = 1 \]  

(24)

Now taking the appropriate derivatives of Eq. (23); results in

\[ \frac{\partial T}{\partial t} = \phi \frac{\partial \phi}{\partial t} ; \quad \frac{\partial T}{\partial r} = \theta \frac{\partial \phi}{\partial r} ; \quad \frac{\partial^2 T}{\partial r^2} = \theta \frac{\partial^2 \phi}{\partial r^2} \]

(25)

Substituting Eq. (25) into Eq. (23) yields:

\[ \frac{\partial \phi}{\partial t} = \alpha \left[ \frac{1}{\phi} \frac{\partial^2 \phi}{\partial r^2} + \frac{1}{r} \frac{\partial \phi}{\partial r} \right] \]

Dividing both sides by \( \phi \theta \) brings:

\[ \frac{1}{\theta} \frac{\partial \theta}{\partial t} = \alpha \left[ \frac{1}{\phi} \frac{\partial^2 \phi}{\partial r^2} + \frac{1}{r} \frac{\partial \phi}{\partial r} \right] = - \kappa \]

(26)

It can be seen that the variables have been separated. Thus, both sides of Eq. (26) must equal a constant, \( \kappa \). Now using the time dependent side of the above, i.e.,

\[ \frac{1}{\theta} \frac{\partial \theta}{\partial t} = - \kappa \]

and multiplying this by \( \theta \phi / \phi \) produces:

\[ \frac{1}{T} \frac{\partial T}{\partial t} = - \kappa \]

Integrating for a particular value of the radial position "r" gives:

\[ T = C_1 \exp \{- \kappa t\} \]
Next the initial condition is used and the equation can be written as:

\[ T(r, t) = T(r, 0) \exp \{-\kappa t\} \quad (27) \]

Returning to Eq. (26) and using the radial side of the equation, we have

\[ \alpha \left[ \frac{1}{r} \frac{\partial^2 T}{\partial r^2} + \frac{1}{r} \frac{\partial T}{\partial r} \right] = -\kappa \]

Multiplying by \( \partial T / \partial r \), the equation can be rewritten as:

\[ \alpha \left[ \frac{1}{T} \frac{\partial^2 T}{\partial r^2} + \frac{1}{r} \frac{\partial T}{\partial r} \right] = -\kappa \quad (28) \]

Incorporating the initial condition,

\[ \frac{\alpha}{T(r, 0)} \left[ \frac{\partial^2 T}{\partial r^2} + \frac{1}{r} \frac{\partial T}{\partial r} \right] = -\kappa \quad (29) \]

Next the partial derivatives are replaced using central finite differences [4]:

\[ \frac{\partial^2 T}{\partial r^2} = \frac{T_{i+1}^n + T_{i-1}^n - 2T_i^n}{(\Delta r)^2} \quad (30) \]

\[ \frac{\partial T}{\partial r} = \frac{T_{i+1}^n - T_{i-1}^n}{2\Delta r} \]

Substituting Eq. (30) into Eq. (29):

\[ \alpha \left[ \left( \frac{T_{i+1}^n + T_{i-1}^n - 2T_i^n}{(\Delta r)^2} \right) + \frac{1}{r_i} \left( \frac{T_{i+1}^n - T_{i-1}^n}{2\Delta r} \right) \right] = -\kappa \quad (31) \]

Equation (31) is used to replace the constant, \(-\kappa\) in Eq. (27), i.e.,

\[ T_{i+1}^{n+1} = T_i^n \exp \left\{ \frac{\alpha \Delta t}{T_i^n} \left[ \left( \frac{T_{i+1}^n + T_{i-1}^n - 2T_i^n}{(\Delta r)^2} \right) + \frac{1}{r_i} \left( \frac{T_{i+1}^n - T_{i-1}^n}{2\Delta r} \right) \right] \right\} \]
Rearranging this brings:

\[ T_1^{n+1} = T_1^n \exp \left\{ \frac{\alpha \Delta t}{(\Delta r)^2} \left[ \left( \frac{T_{1+1}^n + T_{1-1}^n}{T_1^n} - 2T_1^n \right) + \frac{\Delta r}{2r_1} \left( \frac{T_{1+1}^n - T_{1-1}^n}{T_1^n} \right) \right] \right\} \quad (32) \]

Equation (32) states that the temperature at the \( i \)th radial position at the \((n+1)\) time step is found from the product of temperature at \( i \)th position, \( n \)th time step and the exponential term that is composed of a dimensionless time step:

\[ \Omega = \frac{\alpha \Delta t}{(\Delta r)^2} \quad (33) \]

and a dimensionless drive number:

\[ M_1^n = \left( \frac{T_{1+1}^n + T_{1-1}^n}{T_1^n} - 2T_1^n \right) + \frac{\Delta r}{2r_1} \left( \frac{T_{1+1}^n - T_{1-1}^n}{T_1^n} \right) \quad (34) \]

It should be noted that this drive number applies to the interior nodes \((r_1 \neq 0)\). For the node at \( r = 0 \) the dimensionless drive number becomes:

\[ M_1^n = \frac{2(T_{1-1}^n - T_1^n)}{T_1^n} \quad (35) \]

Finally Eq. (32) can be written in a compact form as

\[ T_1^{n+1} = T_1^n \exp \left\{ \Omega M_1^n \right\} \quad (36) \]

The sub-time interval evaluation for Eq. (36) is the same as that found in the two-dimensional Cartesian form as shown earlier.

**Stability of the Exponential Finite Difference Method**

With few exceptions, explicit finite difference procedures for solving partial differential equations are inherently unstable, unless certain numerical conditions are satisfied. These conditions take the
form of a grid size and/or time step requirement written in terms of parameters of the given problem. If these stability conditions are not met, the solution will diverge, often rather drastically. On the other hand, the stability requirements can make explicit methods impractical for a particular application by requiring an unrealistically small grid or time step. Nonetheless, these conditions must be known prior to the use of any explicit finite difference procedure.

There are a variety of methods that have been used to establish the stability constraints of a finite difference procedure: some are very elementary, some quite involved. In essence, the methods seek to find an expression for the amplification factor which is the ratio of the current solution result to that in the previous step. If the absolute value of the ratio is less than one the method is stable. Determination of the amplification factor for the exponential finite difference method is particularly convenient, as has been shown in [2]. For the one-dimensional cylindrical coordinate case, the amplification factor \( \xi \) can be readily defined as

\[
\xi = \frac{T_{1}^{n+1}}{T_{1}^{n}} = \exp \left( \frac{\alpha \Delta t}{(\Delta r)^2} \left[ \frac{(T_{1}^{n+1} + T_{1-1}^{n} - 2T_{1}^{n})}{T_{1}^{n}} \right] + \frac{\Delta r}{2r_{1}} \left( \frac{T_{1+1}^{n} - T_{1-1}^{n}}{T_{1}^{n}} \right) \right)
\]

(37)

or from Eq. (32) for an interior node:

\[
\xi = \frac{T_{1}^{n+1}}{T_{1}^{n}} = \exp \{ \Omega M_{1}^{n} \}
\]

(38)

So for stability to exist as \( \Delta t \) and \( \Delta r \) approach zero:
To satisfy this requirement the exponent in the exponential of Eq. (38) must obviously be less than or equal to zero. Since the components that make up $\Omega$ in that exponent are all positive, this implies that the dimensionless drive number will dictate whether or not the stability criteria is met. For the cylindrical coordinate case the dimensionless drive number must satisfy:

$$\frac{M^n}{T_1} = \left( T^n_{i+1} + T^n_{i-1} - 2T^n_1 \right) + \frac{\Delta r}{2r_1} \left( T^n_{i+1} - T^n_{i-1} \right) \leq 0$$

(40)

Multiplying by $T^n_1$ this becomes:

$$\left( T^n_{i+1} + T^n_{i-1} - 2T^n_1 \right) + \frac{\Delta r}{2r_1} \left( T^n_{i+1} - T^n_{i-1} \right) \leq 0$$

Define $\beta \equiv \Delta r/2r_1$ and rearrange to get

$$(1 + \beta)T^n_{i+1} + (1 - \beta)T^n_{i-1} \leq 2T^n_1$$

$$T^n_1 \geq \frac{1}{2} (1 + \beta)T^n_{i+1} + (1 - \beta)T^n_{i-1}$$

(41)

Equation (41) needs to be satisfied otherwise an unstable condition can exist. In Eq. (41) as $\Delta r \to 0$, or equivalently $\beta \to 0$, the stability condition becomes:

$$T^n_1 \geq \frac{1}{2} T^n_{i+1} + T^n_{i-1}$$

(42)

Also for one-dimensional cylindrical coordinate case, the node at $r = 0$ has a different stability criteria because the node at $i+1$ does not exist. Since the radial derivative of the field variable must equal zero at $r = 0$. In finite difference terms this can be realized...
by requiring that $T_{1+1}^n = T_{1-1}^n$ at this node. Thus, at the origin, Eq. (42) becomes

$$T_1^n \leq T_{1-1}^n$$

(43)

As stated by Bhattacharya [2], the dimensionless drive number is the determining factor whether or not the stability criteria is met. However, the dimensionless time, if made large enough, could cause the solution to become unstable. Since time sub-interval division is used, the total dimensionless time step $\Omega$ could become quite large. From [2] it was recommended that the dimensionless time step be chosen to satisfy the following:

$$\frac{\Omega}{m+1} \leq 0.5$$

(44)

where $m$ is the number of time step sub-intervals involved in the calculations. If $\Omega = 5$, for example, then $m$ would have to be greater than or equal to 9 i.e., nine sub-intervals would be required. From Eq. (21) with infinite heat transfer coefficient, a minimum of 20 nodes would be needed.

Another useful comparison to be made is the one-node model as used in Refs. [2] and [7] where the value of the dependent variable at the surrounding nodes is set equal to zero. For the one-node model, as stated in Ref. [7], the exponential finite difference and exact are the same (Fig. 4). This figure indicates that the exponential solution remains stable as $\Omega$ is increased.
Effect of Initial and Boundary Conditions on the Exponential Finite Difference Method

In this section the effect of boundary conditions on the exponential finite difference technique will be investigated. Boundary conditions that are typical of heat transfer applications will be considered. The conditions to be presented are: (1) finite heat transfer coefficient (mixed condition), (2) infinite heat transfer coefficient (Dirichlet condition), (3) constant heat flux (Neumann condition), and (4) time varying. Initial conditions, where the field variable is equal to zero, will also be discussed.

Finite Heat Transfer Coefficient

For this boundary condition, the method used in reference [2] will be utilized. Numerical implementation of the boundary condition requires that a node be placed outside the solid in the surrounding medium. This external node will be used in the finite difference equation at the solid surface. One-dimensional heat transfer in a cylinder \( T = T(r,t) \) will be used to demonstrate the procedure.

Using the exponential finite difference method for \( T = (r,t) \), it was found earlier (Eq. (36)) that

\[
T_{n+1}^1 = T_1^n \exp \left\{ \Omega M_1^n \right\}
\]  

(45)
where the dimensionless time is given by

\[ \Omega = \frac{\alpha \Delta t}{(\Delta r)^2} \]

and the dimensionless drive number by:

\[
M_1^n = \left\{ \left( \frac{T_{1+1}^n + T_{1-1}^n - 2T_1^n}{T_1^n} \right) + \Delta r \left( \frac{T_{1-1}^n - T_{1+1}^n}{T_1^n} \right) \right\}
\]

The thermal condition at the surface is found by equating the conductive and convective heat fluxes at the surface:

\[ -k \frac{\partial T}{\partial r} \bigg|_{r=R} = h \left( T \bigg|_{r=R} - T_\infty \right) \]

Using the node numbering as shown in Fig. 5 and a central difference to express the derivative, Eq. (47) becomes:

\[ h(T_1^n - T_\infty) = -k \left( \frac{T_0^n - T_2^n}{2 \Delta r} \right) \]

Solving for \( T_0^n \), the temperature of the external node, yields

\[ T_0^n = T_2^n + \frac{2h \Delta r}{k} (T_\infty - T_1^n) \]

Let \( B = h \Delta r/k \) (Biot number [8]), thus Eq. (49) is written:

\[ T_0^n = T_2^n + 2BT_\infty^n - 2BT_1^n \]

Now that an expression for the temperature at the external node is known, it can be used in the expression for the temperature at the surface node. This results in:

\[ T_1^{n+1} = T_1^n \exp \{ \Omega M_1^n \} \]

where

\[
M_1^n = \left\{ \left[ T_2^n + 2BT_\infty + T_2^n - 2BT_1^n \right] \right\} + \frac{\Delta r}{R} \left[ \frac{B(T_\infty^n - T_1^n)}{T_1^n} \right]
\]
and the drive number can be further simplified to:

\[ M^n_1 = \left\{ \left[ \frac{2 T^n_2 - (1 + B)T^n_1 + BT^n_\infty}{T^n_1} \right] + \frac{\Delta T B}{R} \left[ \frac{T^n_\infty - T^n_1}{T^n_1} \right] \right\} \]  \hspace{1cm} (52)

Infinite heat transfer coefficient

If the heat transfer coefficient in Eq. (47) is placed on the left-hand side of the equation and then allowed to become infinite, it is seen that the surface temperature will equal the temperature of the surroundings. Thus, the boundary condition in which \( h \to \infty \) is identical to that in which a boundary temperature is held constant. In the calculation procedure, these isothermal boundary nodes are only needed for calculation of the temperature field at the surrounding nodes. For example in a two-dimensional square grid with a total of 121 nodes calculation would be reduced to a total of 81 nodes if the temperature is specified for all four boundaries.

Constant Heat Flux

For a constant heat flux applied to the boundary surface, the same procedure as was used in the finite heat transfer coefficient case will be utilized. The condition at the surface for one-dimensional cylindrical coordinates is given by:

\[ q = k \frac{\partial T}{\partial r} \bigg|_{r=R} \]  \hspace{1cm} (53)

An external node is placed outside the solid as was done earlier. Equation (53) can then be written

\[ q = k \left( \frac{T^n_0 - T^n_2}{2 \Delta r} \right) \]  \hspace{1cm} (54)
Solving for the external node temperature results in:

\[ T_0^n = \frac{2q \Delta r}{k} + T_2^n \] (55)

At the surface the exponential finite difference equation is:

\[ T_1^{n+1} = T_1^n \exp \left\{ \Omega \left[ \left( \frac{T_0^n + T_2^n - 2T_1^n}{T_1^n} \right) + \frac{\Delta r}{2R} \left( \frac{T_0^n - T_2^n}{T_1^n} \right) \right] \right\} \]

and substituting in Eq. (55):

\[ T_1^{n+1} = T_1^n \exp \left\{ \Omega \left[ \left( \frac{2q \Delta r}{k} + \frac{T_2^n + T_2^n - 2T_1^n}{T_1^n} \right) + \frac{\Delta r}{2R} \left( \frac{2q \Delta r}{k} + \frac{T_2^n - T_2^n}{T_1^n} \right) \right] \right\} \]

Rearranging this produces:

\[ T_1^{n+1} = T_1^n \exp \left\{ \Omega \left[ \frac{2q \Delta r}{k} + \frac{2T_2^n - T_1^n}{T_1^n} + \frac{(\Delta r)^2 q}{Rk T_1^n} \right] \right\} \] (56)

**Time Varying Boundary Conditions**

This condition is similar to the constant boundary temperature condition except that the boundary temperature must be incremented as the calculation marches in time. The boundary condition must reside in the time step loop which is shown as the outer most loop in Fig. 3. The temperatures on these boundaries are incremented and held constant as the subtime interval calculations are made.

**Dependent Variable Initially Equal to Zero**

One last condition that can exist wherever there is initial zero temperature \((T(x,0) = 0)\) needs to be discussed. If this condition is encountered, then the following substitution should be made or else the exponential finite difference method will not work. This can be readily seen by examining any of the numerical equations e.g., Eq. (56). Since the initial temperature would appear in the denominator of the
exponent in the exponential, problems would ensue. To circumvent this difficulty define a new variable $\bar{T}$, such that $\bar{T}(x,t) = 1.0 - T(x,t)$. Now the exponential finite difference equations described above can be utilized by simple replacement of the $T$ variable with the $\bar{T}$ variable.
III. NUMERICAL COMPARISON OF THE EXPONENTIAL FINITE DIFFERENCE METHOD TO EXACT SOLUTIONS AND OTHER NUMERICAL TECHNIQUES

The final product of any numerical study is how well the given method performs when compared to known exact solutions or to other numerical techniques. The exponential finite difference method will now be applied to the following cases to demonstrate the capability of the method to solve the diffusion equation:

(i) One-dimensional heat conduction in cylindrical coordinates with an infinite and a finite heat transfer coefficient at the surface, unsteady state,

(ii) Two-dimensional unsteady state heat conduction in Cartesian coordinates,

(iii) Solution of Laplace's equation, Cartesian coordinates,

(iv) One-dimensional heat conduction in Cartesian coordinates, unsteady state with temperature varying thermal conductivity,

(v) Steady state Couette flow,

(vi) Three-dimensional heat conduction in Cartesian coordinates, unsteady state.

One-Dimensional Heat Conduction in Cylindrical Coordinates

The one-dimensional cylindrical coordinate heat conduction case with temperature as a function of time and radial position will be investigated for infinite and finite heat transfer coefficient. The exact results for both cases can be found in Ref. [9].
For infinite heat transfer coefficient on the boundary surface the exact result is given in [9] as:

\[
\frac{T(r,t) - T_\infty}{T_0 - T_\infty} = 2 \sum_{m=1}^{\infty} e^{-\alpha \lambda_m^2 t} \frac{J_0(\lambda_m r)}{(\lambda_m^2 r^2 + B^2)} \frac{J_1(\lambda_m R)}{J_0(\lambda_m R)} \quad (57)
\]

where \( \lambda_m \) is the \( m \)th zero of \( J_0(\lambda_m R) = 0 \) \( (58) \).

The results of both the exact analysis and the exponential finite difference method are shown in Table I. As can be seen from the tabulated results, exponential finite difference results approach the exact solution as the number of nodes is increased or as the dimensionless time step is decreased.

When the heat transfer coefficient has a finite value at the surface, the exact solution from [9] is:

\[
\frac{T(r,t) - T_\infty}{T_0 - T_\infty} = 2B \sum_{m=1}^{\infty} e^{-\alpha \lambda_m^2 t} \frac{J_0(\lambda_m r)}{(\lambda_m^2 r^2 + B^2)} \frac{J_1(\lambda_m R)}{J_0(\lambda_m R)} \quad (59)
\]

where \( B = hR/k \) (Biot number) and \( \lambda_m \) (characteristics values) are given by (for cooling):

\[
(\lambda_m R)J_1(\lambda_m R) = B J_0(\lambda_m R) = 0 \quad (60)
\]

The results are shown in Table II for various values of the Biot number. As would be expected the solution approaches the exact solution as the number of nodes in increased. The size of the Biot number did not seem to effect the accuracy of the solution. As the elapsed time of the solution proceeded, temperatures predicted by the
exponential finite difference method approached the exact result. Also the results indicated that reducing the size of the time sub-interval increased the method's accuracy.

One last comparison will be made while investigating the exponential finite difference technique in one-dimensional cylindrical coordinates. The problem situation is shown in Fig. (6) and applies to a cylindrical annulus with the following initial and boundary conditions:

\[ T(r,0) = 0 \]
\[ T(R_2,t) = 1.0 \]
\[ \frac{\partial T}{\partial r}(R_1,t) = 0 \]

In Ref. [4] this problem was solved numerically using a characteristic-value solution. A comparison of results is shown in Table III for the exponential method using the same grid spacing as in [4] and for the case where grid spacing is halved. The results are seen to compare quite well with the finer mesh being slightly closer to the value from Ref. [4] especially during the first few time steps of the solution.

Two-Dimensional Heat Conduction in Cartesian Coordinates

The exponential finite difference technique will now be applied to the two-dimensional heat conduction problem in Cartesian coordinates shown in Fig. (7). The exponential finite difference method will be compared to the solution of this problem, as performed in Ref. [4], using the alternating direction implicit technique (ADI). The results of the two numerical techniques and the exact analysis are shown in Table IV. The temperature indicated for comparison is that at the
origin \( x = y = 0 \), shown in Fig. (7). As maybe seen, the ADI technique does not predict the temperature as accurately as the exponential finite difference method at the first time value shown in Table IV. However, as the amount of elapsed time increases either method does a very good job at predicting this temperature. When the number of grid points was increased, by halving the spatial intervals, the exponential finite difference method was found to be more accurate for all the time steps.

Since the ADI method is one that requires simultaneous solution of equations in the two coordinate directions, the time step size can be made large. The exponential finite difference technique must have the dimensionless time step kept below 0.25 to keep the solution stable. So the required CPU time for the exponential method is higher for this application.

**Solution of Laplaces Equation**

Since the exponential finite difference method has been used for two-dimensional unsteady state conduction, a natural extension with little additional effort would be to use this method to solve Laplace's equation. This can be implemented in the exponential finite difference method by just allowing the solution to march in time until no further change in the field variable is indicated.

As an example, the problem as shown in Fig. (8) will be solved and the results compared to those given in Ref. [4]. In the referenced work, the solution was found by using a Gauss-Seidel iterative technique.
A comparison of results along a diagonal from the position \((x = 0,\ y = 1)\) to \((x = 1,\ y = 0)\) is presented in Table V for two different grid spacings. As can be seen, the solutions are nearly identical with the exponential method requiring a smaller number of iterations (or time step increments) to reach a similar result.

One-Dimensional Unsteady State Conduction With Temperature Varying Thermal Conductivity

The effect of temperature varying thermal conductivity will now be investigated using three different numerical schemes: a pure explicit, the exponential method and an implicit technique. The problem to be solved is illustrated in Fig. (9a). The thermal conductivity as shown in Fig. (9b) is assumed to be a linear function of temperature.

The exponential finite difference method will be applied first to the given problem. The governing partial differential equation is [1]:

\[
\rho C_p \frac{\partial T}{\partial t} = \frac{a}{a x} \left( k \frac{\partial T}{\partial x} \right) \tag{62}
\]

From [1], Eq. (62) can be changed to a simpler form by using a new variable \(\theta\) (the Kirchoff transformation) given by:

\[
\theta = \frac{1}{k_R} \int_{T_R}^{T} k(T) dT \tag{63}
\]

where \(k_R\) is the conductivity at temperature \(T_R\), and

\[
\frac{a \theta}{a t} = \frac{k}{k_R} \frac{a T}{a t} \quad \text{or} \quad \frac{a T}{a t} = \frac{k_R}{k} \frac{a \theta}{a t} \tag{64}
\]

Substituting Eq. (64) into (62) gives:
\[
\frac{\rho c_p k R}{k} \left( \frac{\partial \theta}{\partial t} \right) = \frac{\partial}{\partial x} \left( k_R \frac{\partial \theta}{\partial x} \right)
\]

or,

\[
\frac{\rho c_p}{k} \left( \frac{\partial \theta}{\partial t} \right) = \frac{\partial^2 \theta}{\partial x^2}
\]

Since it has been assumed that the thermal conductivity is a linear function of temperature,

\[
k(T) = k_R (1 + \beta T)
\]

Now returning to Eq. (63) and substituting in Eq. (66), we have:

\[
\theta = \frac{1}{k_R} \int_{T_R}^{T} (k_R + \beta k_T) dT
\]

Direct integration yields:

\[
\theta = (T - T_R) \left\{ 1 + \frac{\beta}{2} (T + T_R) \right\}
\]

Equation (67) provides the relationship between the variable \( T \) and the new variable \( \theta \).

Returning to Eq. (65),

\[
\frac{\partial \theta}{\partial t} = \frac{k}{\rho c_p} \frac{\partial^2 \theta}{\partial x^2}
\]

Equation (68) is in a form now that the exponential finite difference can be applied. The resulting equation in the variable can be shown to be given by:

\[
\theta_1^{n+1} = \theta_1^n \exp \left\{ \frac{\Delta t}{\rho c_p (\Delta x)^2} \left[ k_1 \left( \frac{\theta_1^{n+1} + \theta_1^{n-1} - 2\theta_1^n}{\theta_1^n} \right) \right] \right\}
\]

(69)
Evaluating Eq. (67) at node 1 brings
\[ \theta_1^n = (T_1^n - T_R) \left[ 1 + \frac{\beta}{2} (T_1^n + T_R) \right] \]
or
\[ \theta_1^n = (T_1^n - T_R) + \frac{\beta}{2} \left( (T_1^n)^2 + T_R^2 \right) \] (70)

Substitution of Eq. (70) into Eq. (69) at the appropriate time steps and nodal locations will give:
\[ \left( T_{1+1}^n - T_R \right) + \frac{\beta}{2} \left( \left( T_{1+1}^n \right)^2 - T_R^2 \right) = \left[ \left( T_{1-1}^n - T_R \right) + \frac{\beta}{2} \left( (T_{1-1}^n)^2 - T_R^2 \right) \right] \cdot \exp \left\{ \gamma k_1^n \left[ \left( T_{1+1}^n + T_{1-1}^n - 2T_{1_{-1}}^n \right) + \frac{\beta}{2} \left( (T_{1+1}^n)^2 + (T_{1-1}^n)^2 - 2(T_{1_{-1}}^n)^2 \right) \right] \right\} \]
\[ \text{where} \]
\[ \gamma = \frac{\Delta t}{\rho C_p (\Delta x)^2} \]

If \( T_R = T_\infty = 0.0 \), Eq. (71) becomes:
\[ T_{1+1}^n + \frac{\beta}{2} \left( T_{1+1}^n \right)^2 = \left( T_{1-1}^n + \frac{\beta}{2} T_{1_{-1}}^n \right) \cdot \exp \left\{ \gamma k_1^n \left[ \left( T_{1+1}^n + T_{1-1}^n - 2T_{1_{-1}}^n \right) + \frac{\beta}{2} \left( (T_{1+1}^n)^2 + (T_{1-1}^n)^2 - 2(T_{1_{-1}}^n)^2 \right) \right] \right\} \]
(72)

The equation for \( T_{1_{+1}}^{n+1} \) is a quadratic with the right-hand side of the equation all being known at time step \( n \), so define a variable \( \kappa_1 \) such that
\[ \kappa_1 = \left[ T_{1+1}^n + \frac{\beta}{2} (T_{1_{+1}}^n)^2 \right] \exp \left\{ \gamma k_1^n \left[ \frac{(T_{1+1}^n + T_{1_{-1}}^n - 2T_{1_{-1}}^n)}{T_{1+1}^n + \frac{\beta}{2} (T_{1_{+1}}^n)^2} \right. \right. \]
\[ \left. \left. + \frac{\beta}{2} \left[ (T_{1+1}^n)^2 + (T_{1_{-1}}^n)^2 - 2(T_{1_{-1}}^n)^2 \right] \right] \right\} \]
(73)
Equation (72) then becomes:

\[
(T_{ni+1})^2 + \frac{2}{\beta} T_{ni+1} - \frac{2}{\beta} \kappa_1 = 0
\]  

(74)

Solving this and using the positive root results in:

\[
T_{ni+1} = \frac{1}{\beta} \left( -1 + \sqrt{1 + 2\kappa_1 \beta} \right)
\]

(75)

where

\[
\beta > 0
\]

Equation (75) in conjunction with Eq. (74) are implemented in the exponential finite difference solution sequence. In this case the conductivity as well as the temperature field must be kept track of on the sub-time interval level. The dimensionless time step, \( \Omega \), and the rate of conductivity change, \( \beta \), must be both considered when choosing the step size so the solution does not become unstable. For this method, the term \( \gamma k_1^n / (m+1) \) in the exponential was considered at its maximum possible value and the time step was adjusted to retain stability. This criteria was chosen so that

\[
\frac{\gamma k_1^n}{m+1} < 0.5
\]

The next method to be investigated for the temperature varying conductivity problem will be the pure explicit method. As stated earlier the governing partial differential equation for one-dimensional conduction is given by:

\[
\rho C_p \frac{\partial T}{\partial t} = \frac{\partial}{\partial x} \left(k \frac{\partial T}{\partial x}\right)
\]

(76)

Using the chain rule this equation can be put into a nonconservative form.
\[ \rho C_p \frac{dT}{dt} = k \frac{\partial^2 T}{\partial x^2} + \beta \frac{\partial T}{\partial x} \left( \frac{\partial T}{\partial x} \right)^2 \]  

(77)

Assuming the same linear profile as before:

\[
k(T) = k_R + \beta k_R T
\]

then

\[
\frac{\partial k}{\partial t} = \beta k_R
\]

Substituting Eq. (78) into (77) will give:

\[
\rho C_p \frac{dT}{dt} = k_R(1 + \beta T) \frac{\partial^2 T}{\partial x^2} + \beta k_R \left( \frac{\partial T}{\partial x} \right)^2
\]

or

\[
\frac{\partial T}{\partial t} = \alpha_R(1 + \beta T) \frac{\partial^2 T}{\partial x^2} + \beta \alpha_R \left( \frac{\partial T}{\partial x} \right)^2
\]

(78)

where

\[
\alpha_R = \frac{k_R}{\rho C_p}
\]

Using central space differences and a forward time difference we may write:

\[
\frac{\partial T}{\partial x} = \frac{T_{i+1}^n - T_{i-1}^n}{2\Delta x}
\]

\[
\frac{\partial T}{\partial t} = \frac{T_{i+1}^n - T_{i}^n}{\Delta t}
\]

\[
\frac{\partial^2 T}{\partial x^2} = \frac{T_{i+1}^n + T_{i-1}^n - 2T_{i}^n}{(\Delta x)^2}
\]

(79)

Substituting Eq. (79) into (78) produces:

\[
\frac{T_{i+1}^n - T_{i}^n}{\Delta t} = \alpha_R \left( 1 + \beta T_{i}^n \right) \left[ \frac{T_{i+1}^n + T_{i-1}^n - 2T_{i}^n}{(\Delta x)^2} \right] + \beta \alpha_R \left[ \frac{T_{i+1}^n - T_{i-1}^n}{2\Delta x} \right]^2
\]
This may be simplified and written as:

\[ T_1^{n+1} = T_1^n + \Omega \left( \frac{1 + \beta T_1^n}{(1+\beta T_1^n)} \left( T_1^n + T_{1-1}^n - 2T_1^n \right) + \frac{\beta}{4} \left( T_{1+1}^n - T_{1-1}^n \right)^2 \right) \]  

(80)

where

\[ \Omega = \frac{\alpha_R \Delta t}{(\Delta x)^2} \]

Equation (80) can be used to directly solve for the temperature field at the next time step. Some additional care must be used to keep the solution stable as the size of the dimensionless time step \(\Omega\), and the rate of conductivity change, \(\beta\) both effect the solution.

The final method to be implemented for comparative purposes is the implicit method. Starting with Eq. (78), we have:

\[ \frac{\partial T}{\partial t} = \alpha_R \left( 1 + \beta T \right) \frac{\partial^2 T}{\partial x^2} + \beta \alpha_R \left( \frac{\partial T}{\partial x} \right)^2 \]  

(81)

To avoid a solution sequence that would require the solution of nonlinear algebraic equations, the following will be assumed;

(a) The term \((1 + \beta T)\) can be replaced with \((1 + \beta T_1^n)\)

(b) The squared first derivative term can be replaced by

\[ \left( \frac{\partial T}{\partial x} \right)^2 = \left( \frac{T_{1+1}^n - T_{1-1}^n}{2\Delta x} \right) \left( \frac{T_{1+1}^n - T_{1-1}^n}{2\Delta x} \right) \]

This is a linearizing technique known as lagging the coefficients.

Substituting these into Eq. (81) will produce:

\[ \frac{T_1^{n+1} - T_1^n}{\Delta t} = \alpha_R \left( 1 + \beta T_1^n \right) \left[ \frac{T_{1+1}^{n+1} + T_{1-1}^{n+1} - 2T_1^{n+1}}{(\Delta x)^2} \right] \]

\[ + \alpha_R \beta \left( \frac{T_{1+1}^n - T_{1-1}^n}{2\Delta x} \right) \left( \frac{T_{1+1}^n - T_{1-1}^n}{2\Delta x} \right) \]

Further simplification gives:
\[ T_1^{n+1} - T_1^n = \Omega \left( 1 + \beta T_1^n \right) \left( T_{1+1}^{n+1} + T_{1-1}^{n+1} - 2T_1^{n+1} \right) \]
\[ + \frac{\Omega \beta}{4} \left( T_{1+1}^n - T_{1-1}^n \right) \left( T_{1+1}^n - T_{1-1}^n \right) \]  \hspace{1cm} (82)

where

\[ \Omega = \frac{a \Delta t}{(\Delta x)^2} \]

Now define:

\[ \lambda_1^n = \frac{\beta}{4} \left( T_{1+1}^n - T_{1-1}^n \right) \]
\[ \eta_1^n = 1 + \beta T_1^n \]

Substituting Eq. (83) into (82) yields:

\[ T_1^{n+1} - T_1^n = \Omega \left( \eta_1^n \left( T_{1+1}^{n+1} + T_{1-1}^{n+1} - 2T_1^{n+1} \right) \right) + \lambda_1^n \left( T_{1+1}^{n+1} - T_{1-1}^{n+1} \right) \]

Simplifying this results in:

\[ T_1^{n+1} \left( 1 + 2\Omega \eta_1^n \right) - T_{1+1}^{n+1} \left( \Omega \eta_1^n + \Omega \lambda_1^n \right) - T_{1-1}^{n+1} \left( \Omega \eta_1^n - \Omega \lambda_1^n \right) = T_1^n \]  \hspace{1cm} (84)

The equation shown above is now in a form that can be used in the Thomas Algorithm [4], i.e., Eq. (84), can be written as:

\[ a_1 T_{1-1}^{n+1} + b_1 T_1^{n+1} + c_1 T_{1+1}^{n+1} = T_1^n \]

where

\[ a_1 = -\Omega \left( \eta_1^n - \lambda_1^n \right) \]  \hspace{1cm} (85)
\[ b_1 = 1 + 2\Omega \eta_1^n \]
\[ c_1 = -\Omega \left( \eta_1^n + \lambda_1^n \right) \]

Equation (85) can now be solved using a tri-diagonal matrix routine.

The variables \( a_1 \), \( b_1 \), and \( c_1 \) must be evaluated at each position and time step as their values change as the field variable \( T \) changes.
A comparison of results of the three methods can be found in Fig. (10) and Table VI. Figure (10) shows the temperature field through the slab cross-section. From this, it is evident that the exponential and pure explicit methods give very similar results. The implicit method predicted higher temperatures closer to the slab surface and lower temperature at the slab centerline than either of the two explicit methods. In Table VI the results at the slab center are shown for various elapsed times. As can be seen, all three methods agreed with each other to within a few percent.

The Steady State Temperature Field of a Couette Flow

Another application of the exponential finite difference method will now be presented. The problem to be investigated is the developing temperature field in laminar Couette flow [7]. The problem statement is illustrated in Fig. (11). Neglecting viscous effects, the governing equation is given by [10]:

$$\rho C_p U_x \frac{\partial T}{\partial x} = k \left( \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} \right)$$  \hspace{1cm} (86)

Neglecting conduction in the x-direction or assuming that the convection term is much greater than the conduction term, Eq. (86) becomes:

$$U_x \frac{\partial T}{\partial x} = \alpha \frac{\partial^2 T}{\partial y^2}$$  \hspace{1cm} (87)

From Fig. (11) using the expression for the velocity in the x-direction, $U_x = U_y/L$, Eq. (87) becomes
\[ \frac{\partial T}{\partial x} = \alpha x \frac{\partial^2 T}{\partial y^2} \]  

(88)

Equation (88) is now in a form where separation of variables can be implemented. Following the same procedure as indicated earlier to find the exponential finite difference equation, it can be shown that:

\[ T_{j+1}^1 = T_j^1 \exp \left( \frac{\Delta x \alpha L}{U(\Delta y)^2 y_j} \left( \frac{T_{j+1}^1 + T_{j-1}^1}{2T_j^1} - 2T_j^1 \right) \right) \]  

(89)

The procedure utilized here is that the solution marches in the x-direction instead of time as was the case for the previous examples. Information from the last x-position and y-direction are used to determine the dependent variable at the next x-position.

Results of implementing this method are shown in Fig. (12). The temperature field is shown for three x-locations for two different values of L/U. The results indicate that as the upper plate velocity U is increased, the propagation of the temperature change in the y-direction is slowed down.

Unsteady State Heat Conduction in Three-Dimensional Coordinates

The final application of the exponential finite difference method to the diffusion equation will be that of three-dimensional, unsteady state heat conduction. The exponential method, a pure explicit method, and an implicit method (method of Douglas, [11]) will be compared to exact solution for the situation shown in Fig. (13).

The exact solution to the problem illustrated in Fig. 13 is given in Ref. [10] as:
\[
\frac{T(x,y,z,t) - T_1}{T_0 - T_1} = 8 \sum_{m=0}^{\infty} \sum_{n=0}^{\infty} \sum_{p=0}^{\infty} \left( \frac{(-1)^{m+n+p}}{(m+\frac{1}{2})(n+\frac{1}{2})(p+\frac{1}{2})} \right) \cdot \\
\cdot \exp \left\{ - \left( \frac{(m+\frac{1}{2})^2}{a^2} + \frac{(n+\frac{1}{2})^2}{b^2} + \frac{(p+\frac{1}{2})^2}{c^2} \right) \pi^2 \alpha t \right\} \\
\cdot \cos \left[ (m+\frac{1}{2}) \frac{\pi x}{a} \right] \cos \left[ (n+\frac{1}{2}) \frac{\pi y}{b} \right] \cos \left[ (p+\frac{1}{2}) \frac{\pi z}{c} \right]
\]

(90)

where \( a, b, \) and \( c \) are the widths of the cube in the \( x, y, \) and \( z \) directions respectively. Equation (90) will be used to determine how well the numerical techniques predict the temperature distribution.

The exponential finite difference technique will be investigated first. The sequence to be followed for determining the finite difference equation is the same as presented for the earlier cases. The procedure for this three-dimensional case consists of the following stepped procedure:

1. Linearize the partial differential equation
2. Assume a product solution
3. Separate time from spatial dependence
4. Solve for time dependence
5. Insert the appropriate spatial finite differences into exponential term that results from step 3

Based on this procedure the three-dimensional exponential finite difference equation can be shown to be:
\[ T_{n+1}^{1,j,k} = T_{1,j,k}^n \exp \left\{ \Omega \left[ \frac{T_{1+1,j,k}^n + T_{1-1,j,k}^n - 2T_{1,j,k}^n}{T_{1,j,k}^n} \right] \right. \]
\[ + \left. \frac{T_{1,j+1,k}^n + T_{1,j-1,k}^n - 2T_{1,j,k}^n}{T_{1,j,k}^n} \right) + \left( \frac{T_{1,j+1,k}^n + T_{1,j-1,k}^n - 2T_{1,j,k}^n}{T_{1,j,k}^n} \right) \right\} \]

(91)

Using the sub-time interval concept, Eq. (91) becomes:

\[ T_{n+1}^{1,j,k} = T_{1,j,k}^n \exp \left\{ \frac{\Omega}{m+1} \sum_{p=0}^{m} M_{1,j,k}^n p/(m+1) \right\} \]

(92)

where \( m \) is the number of subtime intervals, \( \Omega \) is the dimensionless time step, and \( M_{1,j,k}^n \) is the dimensionless drive number given by.

\[ M_{1,j,k}^n = \left( \frac{T_{1+1,j,k}^n + T_{1-1,j,k}^n - 2T_{1,j,k}^n}{T_{1,j,k}^n} \right) + \left( \frac{T_{1,j+1,k}^n + T_{1,j-1,k}^n - 2T_{1,j,k}^n}{T_{1,j,k}^n} \right) \]
\[ + \left( \frac{T_{1,j+1,k}^n + T_{1,j-1,k}^n - 2T_{1,j,k}^n}{T_{1,j,k}^n} \right) \]

(93)

Equation (92) will be used for all interior nodes in Fig. 13. This equation, as well as those that result from the other analysis, will be adjusted along the insulated boundaries to take into account the boundary condition that exists there.

The next method to be applied to this three-dimensional case will be the pure explicit method. The finite difference equation for this method is given by [11]:

\[ T_{n+1}^{1,j,k} = T_{1,j,k}^n (1 - 6\Omega) + \Omega \left( T_{1+1,j,k}^n + T_{1-1,j,k}^n + T_{1,j+1,k}^n \right) + \Omega \left( T_{1,j-1,k}^n + T_{1,j+1,k}^n + T_{1,j,k-1}^n \right) \]

(94)
where \( \Omega = \frac{\alpha \Delta t}{(\Delta x)^2} \); \( \Delta x = \Delta y = \Delta z \)

As shown in Ref. [11] the dimensionless time step \( \Omega \) must be:

\[
\Omega \leq \frac{1}{6}
\]

(95)
to ensure stability of the method.

The last numerical technique that will be applied is the Method of Douglas [11]. This method is implicit, and the spatial directions are considered sequentially in the \( x, y \), and then \( z \) directions respectively. The intermediate temperatures \( U \) (found from \( x \)-direction sweep) and \( V \) (found from \( y \)-direction sweep) are used to calculate the actual temperature field variable \( T \) (found from \( z \)-direction sweep).

The equations that are solved sequentially are presented as follows.

\[
\frac{U_{1,j,k} - T_{1,j,k}^{n}}{\alpha \Delta t} = \frac{1}{2} \delta^2_x \left( U_{1,j,k} + T_{1,j,k}^{n} \right) + \delta_y \left( T_{1,j,k}^{n} \right) + \delta_z \left( T_{1,j,k}^{n} \right)
\]

(96)

\[
\frac{V_{1,j,k} - T_{1,j,k}^{n}}{\alpha \Delta t} = \frac{1}{2} \delta^2_x \left( U_{1,j,k} + T_{1,j,k}^{n} \right) + \frac{1}{2} \delta^2_y \left( V_{1,j,k} + T_{1,j,k}^{n} \right) + \delta_z \left( T_{1,j,k}^{n} \right)
\]

(97)

\[
\frac{T_{1,j,k}^{n+1} - T_{1,j,k}^{n}}{\alpha \Delta t} = \frac{1}{2} \delta^2_x \left( U_{1,j,k} + T_{1,j,k}^{n} \right) + \frac{1}{2} \delta^2_y \left( V_{1,j,k} + T_{1,j,k}^{n} \right) + \frac{1}{2} \delta^2_z \left( T_{1,j,k}^{n+1} + T_{1,j,k}^{n} \right)
\]

(98)

where the finite difference operator in the \( x \)-direction, for example, would be:

\[
\delta^2_x = \frac{T_{1,j,k}^{n+1} + T_{1,j,k}^{n-1} - 2T_{1,j,k}^{n}}{(\Delta x)^2}
\]

(99)

Equations (96), (97), and (98) must be solved successively because the variable \( U \) is used in equation (97) to find \( V \) and so on. Since the
method operates on one spatial direction at a time, the Thomas Algorithm can be utilized. In the case of finding the \( U \) variable, the \( y \) and \( z \) nodal positions are held constant for all the \( x \)-direction nodal positions (Fig. 14). This process is repeated until all \( y \) and \( z \) nodal values for the \( x \)-direction variable \( U \) are calculated. This procedure is then repeated in a similar way for the \( V \) variable and then finally for the actual temperature field variable.

The results from the three different, three-dimensional solution methods are shown in Table VII. The exponential finite difference method described above outperformed the pure explicit and the method of Douglas for all positions as shown in Table VII.

In Ref. [11] nine different methods to solve the diffusion equation in three dimensions were investigated. The method of Douglas was the preferred method because of its accurate results and low computer CPU time. In that study the pure explicit method required the lowest amount of CPU time with the method of Douglas requiring approximately four times as much. In the present study all three methods were run on two different mainframe computers to investigate how well these three methods compared in CPU times. The results are shown in Table VIII. All three methods were exercised for the same number of time steps. As indicated, the exponential method was approximately three times faster than the method of Douglas but still slower than pure explicit method. From these results it could be concluded that the exponential method would have been chosen as the preferred method had it been used in competition with the nine numerical methods as described in Ref. [11].
IV. - EXTENSION OF THE EXPONENTIAL FINITE DIFFERENCE METHOD TO NONLINEAR PARTIAL DIFFERENTIAL EQUATIONS

The exponential finite difference method will now be applied to two different nonlinear problems. The problems to be addressed will be the viscous Burger's equation and the boundary layer equations (steady state flow over a flat plate).

The viscous Burger's equation is given in Ref. [12] as:

\[
\frac{\partial U}{\partial t} + U \frac{\partial U}{\partial x} = \nu \frac{\partial^2 U}{\partial x^2} \tag{100}
\]

To allow application of the exponential method to this equation, the equation must be first linearized. So letting \( U = A = \text{constant} \), for the nonlinear, term and rearranging the equation, gives:

\[
\frac{\partial U}{\partial t} = -A \frac{\partial U}{\partial x} + \nu \frac{\partial^2 U}{\partial x^2} \tag{101}
\]

Assuming a product solution of the form

\[ U(x,t) = \phi(x)\theta(t) \]

and taking the appropriate derivatives, Eq. (101) becomes

\[
\phi \frac{\partial \theta}{\partial t} = -A \theta \frac{\partial \phi}{\partial x} + \nu \theta \phi \frac{\partial^2 \phi}{\partial x^2}
\]

Division by \( \phi \theta \) gives:

\[
\frac{1}{\theta} \frac{\partial \theta}{\partial t} = -A \frac{\partial \phi}{\partial x} \frac{\phi}{\phi} \frac{\partial^2 \phi}{\partial x^2} = -\kappa = \text{constant} \tag{102}
\]

It can be seen that the terms are now separated. As has been shown earlier, the left-hand side of Eq. (102) can be written as:
\[ U(x,t) = U(x,0) \exp \{- \kappa t \} \] (103)

Now returning to Eq. (102) and examining the \( x \)-dependence, we have

\[ - \frac{A}{\phi} \frac{\partial \phi}{\partial x} + v \frac{\partial^2 \phi}{\partial x^2} = - \kappa \]

Multiplying both sides by \( \phi \) gives:

\[ - A \frac{\partial \phi}{\partial x} + v \phi \frac{\partial^2 \phi}{\partial x^2} = - \kappa \phi \]

This can be written in finite difference form as:

\[ \frac{1}{\nu^n} \left[ - U^n_i \left( \frac{U^n_{i+1} - U^n_{i-1}}{2 \Delta x} \right) + v \left( \frac{U^n_{i+1} + U^n_{i-1} - 2U^n_i}{(\Delta x)^2} \right) \right] = - \kappa \] (104)

This is used to replace the exponent in Eq. (103), thus

\[ U^{n+1}_i = U^n_i \exp \left\{ \frac{\Delta t v}{(\Delta x)^2} \left[ - \frac{\Delta x}{2v} \left( \frac{U^n_{i+1} - U^n_{i-1}}{U^n_i} \right) + \left( \frac{U^n_{i+1} + U^n_{i-1} - 2U^n_i}{U^n_i} \right) \right] \right\} \] (105)

Equation (105) is the exponential finite difference equation for the viscous Burger's equation. An example will now be used to demonstrate the method.

An exact steady state solution to Burger's equation is available for the following conditions

\[ U(0,t) = U_0 \]
\[ U(L,t) = 0 \]

The steady-state solution was given as [12]:

\[ U(x) = U_0 U_1 \left\{ \frac{1 - \exp \left( \frac{0}{R_e} \frac{x}{L} - 1 \right)}{1 + \exp \left( \frac{0}{R_e} \frac{x}{L} - 1 \right)} \right\} \]
where

\[ \text{Re}_L = \frac{U_0 L}{v} \quad (106) \]

and \( U_1 \) is the solution of the equation

\[ \frac{U_1 - 1}{U_1 + 1} = \exp \left\{ - U_1 \text{Re}_L \right\} \]

The exponential finite difference method will be now used to numerically solve the problem stated above. However, for the stated conditions, a problem arises with the portion of the velocity field initially at zero. To overcome this difficulty, the substitution method described earlier will be used. A new variable will be defined such that

\[ \bar{U} = U_0 - U \]

and Burger's equation then becomes:

\[ \frac{\partial \bar{U}}{\partial t} = (\bar{U} - U_0) \frac{\partial \bar{U}}{\partial x} + v \frac{\partial^2 \bar{U}}{\partial x^2} \quad (107) \]

with the following imposed conditions, if \((U_0 = 1)\):

\[ \bar{U}(0,t) = 0 \quad (108) \]

\[ \bar{U}(L,t) = U_0 \]

Using Eq. (107) the same method of separation of variables must be performed on the \( \bar{U} \) variable. The problem is now solved for the \( \bar{U} \) variable and the substitution shown above is then made to find the \( U \) variable. The exponential finite difference equation for \( \bar{U} \) can be shown to be:
\[ 0^n_{1+1} = 0^n_1 \exp \left\{ \frac{\Delta t v}{(\Delta x)^2} \left[ \left( -\frac{\Delta x}{2v} \right) \left[ \frac{1 - 0^n_1}{0^n_1} \left( 0^n_{1+1} - 0^n_{1-1} \right) \right] \right. \right. \\
\left. \left. + \frac{0^n_{1+1} + 0^n_{1-1} - 20^n_1}{0^n_1} \right) \right\} \]  

(109)

The results obtained by applying Eq. (109) and the conditions in Eq. (108) are compared to the steady state exact results of Eq. (106) and are shown in Fig. (15). The results from the exponential method were nearly the same as the exact method. The exponential method was allowed to march in time for quite a number of steps without special treatment of the dimensionless group \( \Delta t v / (\Delta x)^2 \) which could have been altered to allow convergence to the actual solution in less time steps.

Another application of Burger's equation was made to investigate the effect of the diffusion term. The results for the variation of \( v \) over four orders of magnitude are shown in Fig. (16) for the same instant in time. At the two lower \( v \) values, the total range of the field variable takes place over a small number of nodal positions. A better approximation could be made for these cases by using a finer grid. Also included on Fig. (16) is the solution of Burger's equation by a pure explicit technique. For the value of \( v \) chosen, the solution oscillates around the predicted solution found from the exponential method. The pure explicit solution was found using the same number of nodes and the same dimensionless step size. When the solution oscillates, as the pure explicit solution did, the resulting velocity field can contain physically impossible values.
The last application to be investigated will be for the development of a laminar boundary layer on a flat plate (Fig. (17)). In Ref. [10] the steady state formulation is given in terms of the following three partial differential equations:

for continuity:

\[
\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0
\]

(110)

for momentum:

\[
U \frac{\partial u}{\partial x} + V \frac{\partial u}{\partial y} = \nu \frac{\partial^2 u}{\partial y^2}
\]

(111)

for energy:

\[
U \frac{\partial T}{\partial x} + V \frac{\partial T}{\partial y} = \alpha \frac{\partial^2 T}{\partial y^2}
\]

(112)

with the boundary conditions:

\[
U(x,0) = 0 \quad U(0,y) = U_0
\]

\[
V(x,0) = 0 \quad V(x,L) = 0
\]

\[
T(x,0) = 0 \quad T(0,y) = T_0
\]

(113)

\(\nu\) and \(\alpha\) are the momentum and thermal diffusivities respectively.

Equations (111) and (112) can be solved by using the method presented for the viscous Burger's equation. The only difference is that the solution will march in the x-direction instead of time.

Keeping this procedure in mind, results of the separation of variables for Eqs. (111) and (112) were found to be:

\[
U_j^{i+1} = U_j^i \exp \left\{ \frac{\Delta x}{U_j^i} \left[ - \frac{V_j^i}{U_j^i} \left( \frac{U_j^{i+1} - U_j^{i-1}}{2 \Delta y} \right) + \nu \frac{U_j^i \left( U_j^{i+1} + U_j^{i-1} - 2U_j^i \right)}{(\Delta y)^2} \right] \right\}
\]

(114)

\[
T_j^{i+1} = T_j^i \exp \left\{ \frac{\Delta x}{T_j^i} \left[ - \frac{V_j^i}{T_j^i} \left( \frac{T_j^{i+1} - T_j^{i-1}}{2 \Delta y} \right) + \alpha \frac{T_j^i \left( T_j^{i+1} + T_j^{i-1} - 2T_j^i \right)}{(\Delta y)^2} \right] \right\}
\]

(115)
The continuity equation is written as [12]:

\[ V_{j+1}^i = V_{j}^i + \frac{\Delta y}{2 \Delta x} \left( U_{j+1}^i - U_{j}^i + U_{j+1}^{i+1} - U_{j-1}^{i+1} \right) \]  

(116)

Equations (114) and (115) are first solved using a spatial sub-increment as was done for the cases when time was the marching direction of the solution. After this, the continuity Eq. (116), is then solved.

The results of this application are shown in Fig. (18) for a Prandtl number equal to 0.72. As can be seen the thermal boundary layer was outside the velocity boundary layer as would be expected. The results with the Prandtl number equal to 0.72 were compared to the exact solution as presented in Ref. [10]. A downstream position was chosen and the results are compared in Table IX. The exponential method results were in good agreement with the exact results.
CONCLUDING REMARKS

In conclusion, an exponential finite difference technique has been extended to other coordinate systems and expanded to handle problems in two and three dimensions. The method has direct application to linear partial differential equations such as the diffusion equation and can be extended to solve nonlinear equations.

The method was applied to the following cases:

1. One-dimensional, unsteady state heat transfer in cylindrical coordinates, infinite and finite heat transfer coefficient.
2. Two- and three-dimensional, unsteady state heat transfer in Cartesian coordinates.
3. One dimensional heat transfer, with temperature varying thermal conductivity.
4. Developing temperature field in laminar Couette flow.
5. Nonlinear partial differential equations (Burger's equation and boundary layer equations)

The exponential finite difference method predicted the field variable with a higher degree of accuracy in those cases examined where the exact solution was available. When extended to three dimensions, the accuracy was still higher for the exponential finite difference but the computer CPU time was increased. When the exponential method was compared to other numerical techniques, the results were found to be very comparable.
In conclusion, the results predicted for the exponential finite difference algorithm for the cases presented in this study demonstrated that:

(1) Field variable was predicted with a higher degree of accuracy than other numerical techniques where exact solutions exist.

(2) The method can be applied to linear and nonlinear partial differential equations with dependent variables that can be separated.

(3) The stability of the method is the same as that of pure explicit methods, where the sub-time interval step size determines the stability.
REFERENCES


TABLE I. - COMPARISON OF RESULTS FOR DIFFERENT DIMENSIONLESS TIME STEP FOR ONE-DIMENSIONAL HEAT TRANSFER IN CYLINDRICAL COORDINATES WITH INFINITE HEAT TRANSFER COEFFICIENT AT THE SURFACE. INITIAL AND BOUNDARY CONDITIONS ARE:

\[ h \to \infty, \ T(r,o) = 1.0, \ T(R,t) = 0.0, \ \Omega = \alpha \Delta t/\Delta r^2, \ \alpha = 1.0 \text{ M}^2/\text{s}, \]

\[ N = \text{number of nodes, } m = \text{number of sub-time intervals}. \]

<table>
<thead>
<tr>
<th>t, sec</th>
<th>From surface r-distance (M)</th>
<th>N = 11 m = 4 ( \Omega = 1.0 )</th>
<th>N = 21 m = 9 ( \Omega = 1.0 )</th>
<th>N = 21 m = 9 ( \Omega = 2.0 )</th>
<th>N = 21 m = 9 ( \Omega = 5.0 )</th>
<th>Exact analysis ( \text{ref. [9]} )</th>
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<tr>
<td>0.1</td>
<td>0.1</td>
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<td>0.126819</td>
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<td>Total 40 steps</td>
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<tr>
<td>( \alpha = 0.2 )</td>
<td>( \Omega = 0.1 )</td>
<td>( \Omega = 0.2 )</td>
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<td>( \Omega = 5.0 )</td>
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<td>( \frac{m+1}{M+1} = 0.5 )</td>
<td>\ldots</td>
<td>\ldots</td>
</tr>
</tbody>
</table>

TABLE II. - FINITE HEAT TRANSFER COEFFICIENT CYLINDRICAL COORDINATES WITH THE FOLLOWING CONDITIONS: \( T(r,o) = 1.0, \quad T_m = 0, \quad \Omega = \alpha \Delta t/\Delta r^2 \)

| Time | h/k | R | Exact ref. \( \text{ref. [9]} \) | Exponential finite difference results |
|------|-----|---|-----------------|-----------------|-----------------|-----------------|
|      |     |   | \( N = 11 \) m = 4 \( \Omega = 1.0 \) | \( N = 21 \) m = 9 \( \Omega = 1.0 \) | \( N = 21 \) m = 9 \( \Omega = 5.0 \) | \( N = 21 \) m = 9 \( \Omega = 1.0 \) |
| 0.1  | 1   | 1 | 0.6846          | 0.7073          | 0.6978          | \ldots          |
|      | 0   | 1 | 0.9768          | 0.9814          | 0.9797          | \ldots          |
| 0.2  | 1   | 1 | 0.5702          | 0.5976          | 0.5857          | \ldots          |
|      | 0   | 1 | 0.8702          | 0.8852          | 0.8780          | \ldots          |
| 0.4  | 1   | 1 | 0.4132          | 0.4441          | 0.4303          | \ldots          |
|      | 0   | 1 | 0.6420          | 0.6698          | 0.6563          | \ldots          |
| 0.1  | 2   | 1 | 0.5009          | 0.5285          | 0.5199          | \ldots          |
|      | 0   | 1 | 0.9594          | 0.9670          | 0.9643          | \ldots          |
| 0.1  | 5   | 1 | 0.2558          | 0.2777          | \ldots          | 0.2669         |
|      | 0   | 1 | 0.9265          | 0.9385          | \ldots          | 0.9306         |
**TABLE III.** - COMPARISON OF EXPONENTIAL FINITE DIFFERENCE METHOD IN ONE-DIMENSIONAL CYLINDRICAL COORDINATES TO THE RESULTS OF REFERENCE [4].

\[ \alpha = 1.0, \Delta t = 1.0 \text{ sec}, \alpha \Delta t/\Delta r^2 = 1.0. \]

\( N \) = number of nodes, \( m \) = number of sub-intervals

<table>
<thead>
<tr>
<th>Time, sec</th>
<th>( h/k )</th>
<th>( R ), in.</th>
<th>Reference \text{[4]}</th>
<th>( N = 10 )</th>
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<td></td>
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<td>( m = 8 ) ( \Omega = 1.0 )</td>
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<tr>
<td>10</td>
<td>10</td>
<td>18</td>
<td>0.93546</td>
<td>0.935278</td>
</tr>
<tr>
<td>30</td>
<td>10</td>
<td>18</td>
<td>0.57722</td>
<td>0.575979</td>
</tr>
<tr>
<td>90</td>
<td>10</td>
<td>18</td>
<td>0.99370</td>
<td>0.993686</td>
</tr>
<tr>
<td></td>
<td></td>
<td></td>
<td></td>
<td>0.958596</td>
</tr>
</tbody>
</table>

**TABLE IV.** - COMPARISON OF EXPONENTIAL FINITE DIFFERENCE METHOD IN TWO-DIMENSIONAL CARTESIAN COORDINATES TO THE ALTERNATING DIRECTION IMPLICIT METHOD [4].

[For comparison to results in ref. [4] at \( x = y = 0 \).]

<table>
<thead>
<tr>
<th>Time, sec</th>
<th>Exact \text{[4]}</th>
<th>ADI</th>
<th>( \Omega = 1.0, N = 11, \Delta t = 0.01, m = 4 )</th>
<th>( \Omega = 1.0, N = 21, \Delta t = 0.0025, m = 9 )</th>
</tr>
</thead>
<tbody>
<tr>
<td>0.1</td>
<td>0.09883</td>
<td>0.09333</td>
<td>0.09829</td>
<td>0.09924</td>
</tr>
<tr>
<td>.2</td>
<td>.40354</td>
<td>.40354</td>
<td>.40256</td>
<td>.40354</td>
</tr>
<tr>
<td>.3</td>
<td>.63179</td>
<td>.63224</td>
<td>.63080</td>
<td>.63166</td>
</tr>
<tr>
<td>.4</td>
<td>.77486</td>
<td>.77532</td>
<td>.77403</td>
<td>.77472</td>
</tr>
<tr>
<td>.5</td>
<td>.86252</td>
<td>.86283</td>
<td>.86187</td>
<td>.86240</td>
</tr>
<tr>
<td>.6</td>
<td>.91607</td>
<td>.91624</td>
<td>.91559</td>
<td>.91597</td>
</tr>
<tr>
<td>.7</td>
<td>.94877</td>
<td>.94886</td>
<td>.94841</td>
<td>.94869</td>
</tr>
<tr>
<td>.7</td>
<td></td>
<td>(Total of 70 steps)</td>
<td></td>
<td>(Total of 280 steps)</td>
</tr>
</tbody>
</table>

\[ \frac{\Omega}{m + 1} = 0.2 \]

\[ \frac{\Omega}{m + 1} = 0.1 \]
TABLE V. - STEADY STATE HEAT TRANSFER IN TWO-DIMENSIONS

Comparison of exponential finite difference technique to a Gauss-Seidel technique for the solution of Laplace's equation.

<table>
<thead>
<tr>
<th>x</th>
<th>y</th>
<th>9 by 9 Grid</th>
<th>5 by 5 Grid</th>
</tr>
</thead>
<tbody>
<tr>
<td></td>
<td></td>
<td>Gauss-Seidel 88 iterations(a)</td>
<td>Exponential finite difference 40 iterations</td>
</tr>
<tr>
<td>0.000</td>
<td>1.000</td>
<td>0.0</td>
<td>0.0</td>
</tr>
<tr>
<td>.125</td>
<td>.875</td>
<td>1.7413</td>
<td>1.7414</td>
</tr>
<tr>
<td>.250</td>
<td>.750</td>
<td>6.8946</td>
<td>6.8949</td>
</tr>
<tr>
<td>.375</td>
<td>.625</td>
<td>15.0330</td>
<td>15.0335</td>
</tr>
<tr>
<td>.500</td>
<td>.500</td>
<td>24.9999</td>
<td>25.0004</td>
</tr>
<tr>
<td>.625</td>
<td>.375</td>
<td>34.9667</td>
<td>34.9672</td>
</tr>
<tr>
<td>.750</td>
<td>.250</td>
<td>43.1052</td>
<td>43.1055</td>
</tr>
<tr>
<td>.875</td>
<td>.125</td>
<td>48.2587</td>
<td>48.2588</td>
</tr>
<tr>
<td>1.000</td>
<td>0.000</td>
<td>100.0000</td>
<td>100.0000</td>
</tr>
</tbody>
</table>

(a)From reference [4].

TABLE VI. - COMPARISON OF EXPONENTIAL, PURE-EXPLICIT, AND IMPLICIT FINITE DIFFERENCE METHODS FOR ONE-DIMENSIONAL, UNSTEADY-STATE HEAT TRANSFER WITH TEMPERATURE VARYING THERMAL CONDUCTIVITY AT THE CENTER OF THE SLAB

[K(T) = 1.0 + B(T); B = 0.01]

<table>
<thead>
<tr>
<th>Time, sec</th>
<th>Temperature, °C</th>
</tr>
</thead>
<tbody>
<tr>
<td></td>
<td>Exponential finite difference, N = 11, m = 4, Ω = 0.5, Δt = 0.005 sec</td>
</tr>
<tr>
<td>0.01</td>
<td>98.15998</td>
</tr>
<tr>
<td>.02</td>
<td>88.87177</td>
</tr>
<tr>
<td>.05</td>
<td>61.30161</td>
</tr>
<tr>
<td>.1</td>
<td>34.37147</td>
</tr>
</tbody>
</table>
E lapsed
time,
sec

0.09
0.15

Position from
center along
diagonal

Condensed

Exponential finite
\[ T(x,y,z,o) = 1.0; T(x,y,L,t) = T(L,y,z,t) = T(L,y,z,t) = 0; \frac{\partial T}{\partial x}(o,y,z,t) = \frac{\partial T}{\partial y}(o,x,z,t) = \frac{\partial T}{\partial z}(x,y,o,t) = 0; N = \text{number of nodes in } x, y, \text{ and } z \text{ directions}; \Omega = \alpha \Delta t/\Delta x^2 \text{ and } \Delta x = \Delta y = \Delta z. \]

Elapsed
time,
sec

Position from
center along
diagonal

Exact

analysis
result, \( ^\circ C \)

Exponential finite
difference results, \( ^\circ C \)

Pure explicit

finite difference
results, \( ^\circ C \)

Method of Douglas

finite difference
results, \( ^\circ C \)

\[ N = 11, m = 4, \Omega = 0.75 \]

\[ \Omega = 0.15, N = 11 \]

\[ \Omega = 0.15, N = 11 \]

\( a \) Accuracy percent.

<table>
<thead>
<tr>
<th>Elapsed time, sec</th>
<th>Position from center along diagonal</th>
<th>Exact analysis result, (^\circ C)</th>
<th>Exponential finite difference results, (^\circ C)</th>
<th>Pure explicit finite difference results, (^\circ C)</th>
<th>Method of Douglas finite difference results, (^\circ C)</th>
</tr>
</thead>
<tbody>
<tr>
<td>0.09</td>
<td>0.0</td>
<td>0.893490</td>
<td>0.892237 (0.14)</td>
<td>0.889437 (0.45)</td>
<td>0.886760 (0.75)</td>
</tr>
<tr>
<td></td>
<td>.5</td>
<td>0.440712</td>
<td>0.440650 (0.014)</td>
<td>0.435058 (1.28)</td>
<td>0.439665 (0.24)</td>
</tr>
<tr>
<td></td>
<td>.9</td>
<td>0.006491</td>
<td>0.006484 (0.11)</td>
<td>0.006319 (2.65)</td>
<td>0.006510 (0.29)</td>
</tr>
<tr>
<td>0.15</td>
<td>0.0</td>
<td>0.645469</td>
<td>0.645209 (0.04)</td>
<td>0.640025 (0.84)</td>
<td>0.641484 (0.62)</td>
</tr>
<tr>
<td></td>
<td>.5</td>
<td>0.253065</td>
<td>0.253286 (-0.09)</td>
<td>0.250102 (1.17)</td>
<td>0.252641 (0.17)</td>
</tr>
<tr>
<td></td>
<td>.9</td>
<td>0.003015</td>
<td>0.003022 (-0.23)</td>
<td>0.002970 (1.49)</td>
<td>0.003023 (-0.27)</td>
</tr>
</tbody>
</table>

\( a \) Accuracy percent.

<table>
<thead>
<tr>
<th>Computer</th>
<th>Exponential method, sec</th>
<th>Method of Douglas, sec</th>
<th>Pure-explicit method, sec</th>
</tr>
</thead>
<tbody>
<tr>
<td>CRAY-XMP</td>
<td>0.2778</td>
<td>0.955</td>
<td>0.0627</td>
</tr>
<tr>
<td>IBM-3033</td>
<td>5.4</td>
<td>12.6</td>
<td>1.8</td>
</tr>
</tbody>
</table>

\( a \) Based on the number of sub-time intervals equal to 100.
TABLE IX. - COMPARISON OF EXPONENTIAL FINITE DIFFERENCE METHOD TO EXACT RESULTS OF BOUNDARY LAYER EQUATION [10] FOR THE VELOCITY PROFILE AT ONE DOWNSTREAM LOCATION.

[Distance downstream \( x = 500 \) cm, \( v = 0.0072 \) \( \text{cm}^2/\text{s} \)]

<table>
<thead>
<tr>
<th>Distance perpendicular to plate, ( y ) (cm)</th>
<th>Exact result ([10])</th>
<th>Exponential method result, ( N = 21 ), ( m = 8 )</th>
</tr>
</thead>
<tbody>
<tr>
<td>1.0</td>
<td>0.17</td>
<td>0.17428</td>
</tr>
<tr>
<td>2.0</td>
<td>.34</td>
<td>.34643</td>
</tr>
<tr>
<td>3.0</td>
<td>.51</td>
<td>.51020</td>
</tr>
<tr>
<td>4.0</td>
<td>.65</td>
<td>.65658</td>
</tr>
<tr>
<td>5.0</td>
<td>.78</td>
<td>.77684</td>
</tr>
<tr>
<td>6.0</td>
<td>.87</td>
<td>.86636</td>
</tr>
<tr>
<td>7.0</td>
<td>.93</td>
<td>.92638</td>
</tr>
<tr>
<td>8.0</td>
<td>.96</td>
<td>.96265</td>
</tr>
</tbody>
</table>
FIGURE 1.- COMPUTATIONAL GRID FOR EXPONENTIAL FINITE DIFFERENCE TECHNIQUE FOR 2-DIMENSIONAL CARTESIAN COORDINATES.
Figure 2. - Computational grid for 2 time sub-intervals \((M = 2)\), Cartesian coordinates. (shown for 1-dimension).
BEGIN
INITIALIZE
PROGRAM
VARIABLES

L=1...NTOT

L > NTOT
STOP

i=1...N
j=1...N

p_{i,j} = 0.0
V_{i,j} = V_{i,j}

K=1,2...NS1

J=1,2...N
I=1,2...N

M_{i,j} = \frac{V_{i,j} + V_{i-1,j} + V_{i,j-1} + V_{i-1,j-1} - 4V_{i,j}}{V_{i,j}}

I = 1,2,...,N
J = 1,2,...,N

V_{i,j} = V_{i,j} (\exp (\Omega d_{1}/NS1))

FIGURE 3. - FLOW DIAGRAM FOR 2-DIMENSIONAL EXPONENTIAL
FINITE Difference ALGORITHM.

NTOT - TOTAL NUMBER OF TIME STEPS
P_{i,j} - SUM OF DIMENSIONLESS DRIVE NUMBERS
V_{i,j} - FIELD VARIABLE DURING TIME STEP
M_{i,j} - DIMENSIONLESS DRIVE NUMBER
NS1 - NUMBER OF TIME SUB-INTERVALS
\Omega - DIMENSIONLESS TIME \quad (x)^2
<table>
<thead>
<tr>
<th>METHOD</th>
<th>$T_{n+1}/T_n$</th>
</tr>
</thead>
<tbody>
<tr>
<td>Euler (Explicit)</td>
<td>$1 - 2\Omega$</td>
</tr>
<tr>
<td>Crank-Nicolson</td>
<td>$(1 - \Omega)/(1 + \Omega)$</td>
</tr>
<tr>
<td>Pure Implicit</td>
<td>$1/(1 + 2\Omega)$</td>
</tr>
<tr>
<td>Bhattacharya</td>
<td>$\exp(-2\Omega)$</td>
</tr>
</tbody>
</table>

$\Omega = \frac{\Delta t}{(\Delta x)^2}$

**Figure 4.** Effect of nondimensional time step size on 1 node model solution. (From Ref. 21.)
SUBSURROUNDING FLUID

SOLID

FiguRE 5. - EXPONENTIAL FINITE DIFFERENCE NODAL CONFIGURATION FOR 1-DIMENSIONAL HEAT TRANSFER WITH FINITE HEAT TRANSFER COEFFICIENT.
INITIAL CONDITION: \( T(r,0) = 0 \)

BOUNDARY CONDITIONS:
- \( T(R_2, t) = 1.0 \)
- \( \frac{\partial T}{\partial r} (R_1, t) = 0 \)

\( R_1 = 10.0 \) in. \( R_2 = 19.0 \) in.

**Figure 6. Problem Conditions for Comparison of Exponential Finite Difference Technique to Characteristic Problem Solution.**
I.C. : \( f(x, y, 0) = 0.0 \)
R.C.'s: \( f(x, 1.0, t) = f(1.0, y, t) = 1.0 \)
\[
\frac{\partial T}{\partial x} \mid (0, y, t) = \frac{\partial T}{\partial y} \mid (x, 0, t) = 0.0
\]

**Figure 7.** Problem description for 2 dimensional comparison of exponential finite difference method to the alternating direction implicit (ADI) method.
Figure 8. - Problem sketch for comparison of exponential finite difference solution to the Gauss-Seidel iterative method for the solution of Laplace's equation in 2-dimensions.
(A) ONE-DIMENSIONAL PROBLEM WITH VARYING THERMAL CONDUCTIVITY.

$$I(x, 0) = 100.0$$
$$I(0, t) = I(1, t) = 0.0$$

$$k(t) = k_R + \beta k_R^T$$

(B) LINEAR RELATIONSHIP BETWEEN CONDUCTIVITY AND TEMPERATURE.

FIGURE 9. SKETCHS SHOWING PROBLEM STATEMENT FOR TEMPERATURE VARYING THERMAL CONDUCTIVITY.
Figure 10. Comparison of methods for temperature-varying conductivity, showing temperature field at $t = 0.02$ sec. 

**K(T) = k_r (1 + $\beta$), where $k_r = 1.0$, $\beta = 0.01$; $T(x,0) = 100$; $T(0,1) = T(L,1) = 0$; $0 = 1$.**

**Exponential:** $m = 4$; $\Omega = 0.5$; 20 time steps.

**Pure Implicit:** $\Omega = 0.25$; 8 time steps.
MOVING WALL VELOCITY = U, TEMPERATURE = T₀

VELOCITY PROFILE \( u_x = \frac{U}{L} y \)

STATIONARY WALL BOUNDARY CONDITIONS:
\[ T(y,0) = T₀ \quad x < 0 \]
\[ T(0,x) = T₁ \quad x \geq 0 \]

FIGURE 11. - SKETCH SHOWING CONDITIONS FOR DEVELOPING TEMPERATURE FIELD IN LAMINAR COUETTE FLOW.
Figure 12. Developing temperature field in laminar Couette flow, showing the effect of fluid velocity \( U \); \( L = 1.0 \text{ m} \).
3-DIMENSIONAL UNSTEADY STATE CONDUCTION

INITIAL CONDITION: $T(x,y,z,0) = T_0 = 1$

BOUNDARY CONDITIONS: $t > 0$

$\frac{\partial T}{\partial x}(0,y,z,1) = \frac{\partial T}{\partial y}(x,0,z,1) = \frac{\partial T}{\partial z}(x,y,0,1) = 0$

$T(1,y,z,1) = T(x,1,z,1) = T(x,y,1,1) = 0$

FIGURE 13. - BOUNDARY AND INITIAL CONDITIONS FOR THREE DIMENSIONAL UNSTEADY STATE CONDUCTION HEAT TRANSFER.
FIGURE 14. METHOD OF DOUGLAS SHOWING THE PROCEDURE USED TO SWEEP IN THE SUCCESSIVE COORDINATE DIRECTIONS.

EXPONENTIAL FINITE DIFFERENCE RESULT

EXACT ANALYSIS

EXPONENTIAL METHOD PARAMETERS:

\[ N = 21; M = 8; \Delta x = 4.0 \]

\[ v = 0.4 \text{ m}^2/\text{sec}; t = 2.0 \text{ sec shown} \]

FIGURE 15. COMPARISON OF STEADY STATE SOLUTIONS COMPARING THE EXACT RESULTS TO THE EXPONENTIAL FINITE DIFFERENCE SOLUTION \( U(0,t) = 1.0; U(L,t) = 0.0 \).
FIGURE 16. EXPLICIT FINITE DIFFERENCE RESULTS FOR VARYING KINEMATIC VISCOSITY. ALL VELOCITIES ARE SHOWN FOR T = 1.0 SECONDS WITH: N = 21, M = 8, \( \frac{\Delta t}{(\Delta x)^2} = 4.0 \).

\[
\begin{align*}
U(0, t) &= 1.0 \\
U(L, t) &= 0.0 \\
\end{align*}
\]

FIGURE 17. BOUNDARY LAYER DEVELOPMENT ALONG A COOLED FLAT PLATE.
Figure 18. - Exponential finite difference results for boundary layer equations, with conditions $u(0,y) = 1.0$, $u(x,0) = 0$, $v(x,0) = 0$, $T(x,0) = 0.0$, $T(0,y) = 1.0$; $v = 0.0072 \text{ cm/sec}$; $\alpha = 0.01 \text{ cm}^2/\text{sec}$. 

- Velocity boundary layer
- Thermal boundary layer
- Temperature profiles
- Velocity profiles $u = 1$
APPENDIX

This appendix contains all the computer programs mentioned in this report. A computer program variable list is also contained with a description of their use, and a program number to refer to the programs that they are contained in.

Each of these programs was written to be run in an interactive mode with the mainframe computer. The only cases run differently were for the three-dimensional unsteady state heat transfer cases that were run in batch mode on the Cray X-MP.

The program structure is as follows. A main program is used to describe the necessary parameters and for asserting the proper boundary conditions. The main program then calls the subroutine where the actual finite difference methods are exercised and the results are then printed.
<table>
<thead>
<tr>
<th>Program number</th>
<th>Program name</th>
<th>Program function</th>
</tr>
</thead>
<tbody>
<tr>
<td>1</td>
<td>SOURCE.EFDCYL</td>
<td>One-dimension, unsteady state, cylindrical coordinates, infinite and finite heat transfer coefficient</td>
</tr>
<tr>
<td>2</td>
<td>SOURCE.EFD2D</td>
<td>Two-dimensional Cartesian coordinates, unsteady state heat transfer</td>
</tr>
<tr>
<td>3</td>
<td>SOURCE.LAPLAC</td>
<td>Two-dimensional Laplace's equation</td>
</tr>
<tr>
<td>4</td>
<td>SOURCE.EFDVAR</td>
<td>One-dimensional unsteady state heat conduction, varying thermal conductivity, exponential finite difference method</td>
</tr>
<tr>
<td>5</td>
<td>SOURCE.EXPVAR</td>
<td>One-dimensional, unsteady state heat conduction, varying thermal conductivity, explicit finite difference method</td>
</tr>
<tr>
<td>6</td>
<td>SOURCE.IMPVAR</td>
<td>One-dimensional, unsteady state heat transfer, varying thermal conductivity, implicit finite difference method</td>
</tr>
<tr>
<td>7</td>
<td>SOURCE.COUE</td>
<td>One-dimensional, developing temperature field in laminar couette flow</td>
</tr>
<tr>
<td>8</td>
<td>SOURCE.EX3D</td>
<td>Exact analysis, three-dimensional heat transfer in a cube</td>
</tr>
<tr>
<td>9</td>
<td>SOURCE.EFD3D</td>
<td>Three-dimensional unsteady state heat transfer in a cube using exponential finite difference method</td>
</tr>
<tr>
<td>10</td>
<td>SOURCE.EXPL3D</td>
<td>Three-dimensional unsteady state heat transfer in a cube using explicit finite difference method</td>
</tr>
<tr>
<td>11</td>
<td>SOURCE.DOUGLA</td>
<td>Three-dimensional unsteady state heat transfer in a cube using the method of Douglas</td>
</tr>
<tr>
<td>12</td>
<td>SOURCE.BURGER</td>
<td>Exponential solution of nonlinear viscous Burger's equation</td>
</tr>
<tr>
<td>13</td>
<td>SOURCE.EXBURG</td>
<td>Pure explicit solution of nonlinear viscous Burger's equation</td>
</tr>
<tr>
<td>14</td>
<td>SOURCE.NONBOU</td>
<td>Exponential Method of solution for boundary layer equations for flow over a flat plate</td>
</tr>
</tbody>
</table>
### COMPUTER PROGRAM VARIABLE LIST

<table>
<thead>
<tr>
<th>Program variable name</th>
<th>Programs used in</th>
<th>Variable Description</th>
</tr>
</thead>
<tbody>
<tr>
<td>N</td>
<td>1-14</td>
<td>Number of nodes</td>
</tr>
<tr>
<td>NS</td>
<td>1-12,14</td>
<td>Number of time sub intervals</td>
</tr>
<tr>
<td>NTOT</td>
<td>1-14</td>
<td>Total number of time or spatial steps</td>
</tr>
<tr>
<td>TSI</td>
<td>1-13</td>
<td>Dimensionless time step increment</td>
</tr>
<tr>
<td>T</td>
<td>1-7,9,10,12,13</td>
<td>Total elapsed time or spatial distance between steps</td>
</tr>
<tr>
<td>DL</td>
<td>1</td>
<td>Radial distance between adjacent nodes</td>
</tr>
<tr>
<td>R</td>
<td>1</td>
<td>Radial length</td>
</tr>
<tr>
<td>IPR</td>
<td>1-14</td>
<td>Number of steps between output of results</td>
</tr>
<tr>
<td>NB</td>
<td>1</td>
<td>Heat transfer boundary condition flag</td>
</tr>
<tr>
<td>V</td>
<td>1-14</td>
<td>Dependent variable</td>
</tr>
<tr>
<td>HK</td>
<td>1</td>
<td>Convection heat transfer coefficient divided by thermal conductivity</td>
</tr>
<tr>
<td>VM</td>
<td>1</td>
<td>Dependent variable value outside of solid in the surrounding medium</td>
</tr>
<tr>
<td>M</td>
<td>1-10,12</td>
<td>Dimensionless drive number</td>
</tr>
<tr>
<td>P</td>
<td>1-10,12</td>
<td>Sum of the drive numbers</td>
</tr>
<tr>
<td>VT</td>
<td>1-14</td>
<td>Dependent variable value during sub-time interval</td>
</tr>
<tr>
<td>B</td>
<td>1</td>
<td>Biot number</td>
</tr>
<tr>
<td>THE</td>
<td>1-3,12-14</td>
<td>Variable used for the output of results</td>
</tr>
<tr>
<td>TIME</td>
<td>1-6,8-13</td>
<td>Total elapsed time of the solution at the current output</td>
</tr>
<tr>
<td>ITMAX</td>
<td>1-14</td>
<td>Output counter</td>
</tr>
<tr>
<td>ETI</td>
<td>3</td>
<td>Accuracy desired in solution of Laplace's equation</td>
</tr>
<tr>
<td>Program variable name</td>
<td>Program numbers used in</td>
<td>Description</td>
</tr>
<tr>
<td>-----------------------</td>
<td>-------------------------</td>
<td>-------------</td>
</tr>
<tr>
<td>DELV</td>
<td>3</td>
<td>Difference in dependent variable value from one time step to the next</td>
</tr>
<tr>
<td>ETA</td>
<td>3</td>
<td>Sum of the absolute value of the differences found in DELV</td>
</tr>
<tr>
<td>KR</td>
<td>4-6</td>
<td>Reference thermal conductivity</td>
</tr>
<tr>
<td>BETA</td>
<td>4-6</td>
<td>Slope of thermal conductivity variation with temperature</td>
</tr>
<tr>
<td>KO,K</td>
<td>4</td>
<td>Thermal conductivity at the total time step interval or sub-time interval respectively</td>
</tr>
<tr>
<td>THE</td>
<td>4</td>
<td>Kirchoff transformation variable (used in exponential finite difference program with varying thermal conductivity)</td>
</tr>
<tr>
<td>KAPPA</td>
<td>4</td>
<td>All values known from the last time step increment and used to solve the quadratic equation that results in the exponential finite difference solution with temperature varying thermal conductivity</td>
</tr>
<tr>
<td>DERSQR</td>
<td>5</td>
<td>Absolute value of velocity difference found in evaluation of velocity gradient</td>
</tr>
<tr>
<td>OMEGA</td>
<td>6</td>
<td>Same as nondimensional time step</td>
</tr>
<tr>
<td>A,B,C,D</td>
<td>6,11</td>
<td>Coefficient used in tridiagonal matrix algorithm.</td>
</tr>
<tr>
<td>KAP,GAM</td>
<td>6</td>
<td>Variables used to determine A,B,C</td>
</tr>
<tr>
<td>BETA,GAMMA</td>
<td>6</td>
<td>Variables used in Thomas, tridiagonal algorithm</td>
</tr>
<tr>
<td>TS</td>
<td>6</td>
<td>Same as total elapsed time</td>
</tr>
<tr>
<td>T</td>
<td>6</td>
<td>Dependent variable</td>
</tr>
<tr>
<td>V</td>
<td>6</td>
<td>Solution vector tri-diagonal algorithm</td>
</tr>
<tr>
<td>SP</td>
<td>7</td>
<td>Maximum width divided by maximum velocity</td>
</tr>
<tr>
<td>FL</td>
<td>7</td>
<td>Parameter based on position in flow</td>
</tr>
<tr>
<td>DIST</td>
<td>7</td>
<td>Serves same function as time for unsteady state problem</td>
</tr>
<tr>
<td>Program variable name</td>
<td>Program numbers used in</td>
<td>Description</td>
</tr>
<tr>
<td>----------------------</td>
<td>------------------------</td>
<td>-------------</td>
</tr>
<tr>
<td>PI, PI2, PI3</td>
<td>8</td>
<td>( \pi, \pi^2, \pi^3 )</td>
</tr>
<tr>
<td>NODES</td>
<td>8</td>
<td>Same as ( N )</td>
</tr>
<tr>
<td>TO</td>
<td>8</td>
<td>Initial temperature</td>
</tr>
<tr>
<td>TI</td>
<td>8</td>
<td>Surface temperature, ( t &gt; 0 )</td>
</tr>
<tr>
<td>ALFA</td>
<td>8</td>
<td>Thermal diffusivity</td>
</tr>
<tr>
<td>TNEW</td>
<td>8</td>
<td>Exact temperature at a ( x, y, ) and ( z ) location after elapsed time ( t ) has occurred</td>
</tr>
<tr>
<td>T</td>
<td>11</td>
<td>Dependent variable</td>
</tr>
<tr>
<td>DELX, DELY, DELZ</td>
<td>11</td>
<td>Part of the central difference operator</td>
</tr>
<tr>
<td>U, V</td>
<td>11</td>
<td>Variables used to sweep preliminary solution in the ( x ) then ( y ) directions respectively</td>
</tr>
<tr>
<td>M</td>
<td>11</td>
<td>Used as an array to contain the known quantities used in the tridiagonal algorithm</td>
</tr>
<tr>
<td>RNU</td>
<td>12-14</td>
<td>Kinematic viscosity</td>
</tr>
<tr>
<td>DX</td>
<td>14</td>
<td>Step length in flat plate direction</td>
</tr>
<tr>
<td>RAL</td>
<td>14</td>
<td>Thermal diffusivity</td>
</tr>
<tr>
<td>YMAX</td>
<td>14</td>
<td>Maximum distance perpendicular to flat plate</td>
</tr>
<tr>
<td>DY</td>
<td>14</td>
<td>Step length perpendicular to the flat plate</td>
</tr>
<tr>
<td>U, V</td>
<td>14</td>
<td>Dependent variables (velocity) in ( x ) and ( y ) directions respectively</td>
</tr>
<tr>
<td>T</td>
<td>14</td>
<td>Temperature field variable</td>
</tr>
<tr>
<td>MU, MT</td>
<td>14</td>
<td>Drive numbers for velocity in ( x )-direction and temperature field</td>
</tr>
<tr>
<td>PU, PT</td>
<td>14</td>
<td>Sum of drive numbers for ( MU, MT )</td>
</tr>
<tr>
<td>UT, TT, VT</td>
<td>14</td>
<td>( x )-direction velocity, temperature, and ( y )-direction velocity on subinterval</td>
</tr>
<tr>
<td>UTI</td>
<td>14</td>
<td>Temporary ( U )-direction velocity field</td>
</tr>
<tr>
<td>TS, TSI</td>
<td>14</td>
<td>Dimensionless time step for temperature and ( x )-direction velocity respectively</td>
</tr>
</tbody>
</table>
WRITTEN BY R.F. HANDSCHUH

SOURCE.EFDCYL

***** PROGRAM #1 *****

THIS PROGRAM IS TO BE USED AS THE STARTING POINT FOR INVESTIGATING THE EXPONENTIAL FINITE DIFFERENCE ALGORITHM. THIS METHOD WAS INTRODUCED BY M.C. BHATTACHARYA. THIS SOURCE CODE IS FOR CYLINDRICAL COORDINATES, UNSTEADY-STATE HEAT CONDUCTION, 1 DIMENSION.

IMPLICIT REAL*8(A-H,O-Z)
REAL*8 V(100),R(100)

INPUT THE PROGRAM DATA

WRITE(6,15)
15 FORMAT(1X,'NUMBER OF NODES=N I3')
READ(9,10)N
WRITE(6,12)
12 FORMAT(1X,'NUMBER OF TIME SUB INTERVALS= NS I3')
READ(6,13)NS
WRITE(6,16)
16 FORMAT(1X,'TOTAL NUMBER OF TIME STEPS= NTOT I3')
READ(9,21)NTOT
WRITE(6,24)
24 FORMAT(1X,'INPUT TIME*THermal DIFFUSIVITY / RAD INT SQUARED F5.3')
READ(9,25)TSI
25 FORMAT(F5.3)
WRITE(6,22)
22 FORMAT(1X,'TOTAL TIME OF ONE TIME STEP= T F7.5')
READ(9,23)T
23 FORMAT(F7.4)
WRITE(6,26)
26 FORMAT(1X,'INPUT RADIAL INTERVAL LENGTH=DL F5.3')
READ(9,27)DL
27 FORMAT(F5.3)
DO 28 I=2,N
R(I)=R(I-1)+DL
28 R(I)=R(I)+DL
WRITE(6,32)
32 FORMAT(1X,'INPUT NUMBER OF TIME STEPS BEFORE PRINTING I3')
READ(9,33)IPR
33 FORMAT(I3)

DETERMINE THE TYPE OF BOUNDARY CONDITION, THEN SET VALUES
WRITE(6,14)
14 FORMAT(1X,'INPUT HEAT TRANSFER B.C. 0 - INFINITE 1 - FINITE/')
READ(9,17)NB
17 FORMAT(I1)
IF(NB.EQ.1) GO TO 100
V(1)=0.
DO 30 I=2,N
30 V(I)=1.0
CALL EXP FIN DIF FOR INFINITE HEAT TRANSFER COEFFICIENT
CALL EFDFIEC(HK,N,VM,DL,NS,V,NTOT,TSI,T,R,IPR)
GO TO 101
CONTINUE
STOP
END
SUBROUTINE EFDFIEC(N,NS,NTOT,TSI,V,T,R,DL,IPR)
FOR INFINITE HEAT TRANSFER COEFFICIENT
IMPLICIT REAL*8(A-H,O-Z)
REAL*8 VT(100),V(100),M(100),P(100),R(100),THE(100)
TS=TSI/DFLOAT(NS+l)
WRITE(6,21)(R(I),I=1,N)
21 FORMAT(1X,11(F6.3,2X))
WRITE(6,22)DL,TSI
22 FORMAT(1X,2(F6.3,2X))
N1=N-1
NS1=NS+1
BEGIN MAIN TIME STEP LOOP
DO 20 L=1,NTOT
ZERO DRIVE NUMBERS AND SET TEMPORARY VARIABLES EQUAL TO THE
LAST TOTAL TIME STEP VALUES
DO 15 I=1,N
15 P(I)=0.
DO 10 I=1,N
VT(I)=V(I)

SUB TIME INTERVAL

DO 30 K=1,NS1

CALCULATE THE DRIVE NUMBERS

DO 40 I=2,N1
IM1=I-1
IP1=I+1
M(I)=2.*(VT(I)-VT(IM1)-VT(IP1))/VT(I)
40  M(I)=M(I)+DI*(VT(IP1)-VT(IM1))/VT(I)*2.*R(I))
M(N)=2.*(VT(N)-VT(N1))/VT(N)

CALCULATE THE DEPENDENT VARIABLE ON THE SUB-INTERVAL LEVEL

DO 50 I=2,N
VT(I1)=VT(I+1)*DEXP(-TS*M(I1))

SUM THE DRIVE NUMBERS

DO 60 I=2,N
P(I)=P(I)+M(I)
30 CONTINUE

CALCULATE THE DEPENDENT VARIABLE ON THE NEXT COMPLETE TIME STEP

DO 70 I=1,N
V(I)=V(I)*DEXP(-TS*P(I))
70 CONTINUE

ITMAX=ITMAX+1

PRINT THE RESULTS

IF(ITMAX.LT.IPR)GO TO 20
DO 71 I=1,N
THE(I)=V(I)
ITMAX=0
5 FORMAT(/)
WRITE(6,5)
31 FORMAT(1X,'TIME STEP NUMBER=',I3)
TIME=T*DFLOAT(L)
WRITE(6,32)TIME
32 FORMAT(1X,'ELAPSED TIME=',F10.4,' SECONDS')
IF(N.GT.11)N21=N/2
IF(N.GT.11)GO TO 81
WRITE(6,82)(THE(I),I=1,N)
82 FORMAT(11(2X,F8.6))
GO TO 84
81 CONTINUE

WRITE(6,82)(THE(I),I=1,N21)
SUBROUTINE EFDFHC

SUBROUTINE EFDFHC(HK,N,VM,DL,NS,V,NTOT,TSI,T,R,IPR)
IMPLICIT REAL*8 (A-H,O-Z)
REAL*8 VT(100),V(100),M(100),P(100),R(100),THE(100)
B=HK*DL
TS=TSI/DFLOAT(NS+1)
NS1=NS+1
N1=N-1
N2=N-2

BEGIN THE TOTAL TIME STEP LOOP
DO 20 L=1,NTOT

ZERO THE DRIVE NUMBERS AND SET THE TEMPOARY DEPENDENT VARIABLES
EQUAL TO THE LAST COMPLETE STEP VALUES
DO 15 I=1,N
P(I)=0.
DO 10 I=1,N
VT(I)=V(I)

SUB TIME INTERVAL
DO 30 K=1,NS1
CALCULATE THE DRIVE NUMBERS
M(1)=-(2.*VT(2)-(2.*B)*VT(1)+2.*B*VM)/VT(1)
M(1)=M(1)-DL*B*(VT(1)-VM)/(R(1)*VT(1))
DO 40 I=2,N
IM1=I-1
IP1=I+1
M(I)=(2.*VT(I)-VT(IM1)-VT(IP1))/VT(I)
M(I)=M(I)+DL*(VT(IP1)-VT(IM1))/(VT(I)*2.*R(I))
M(N)=2.*(VT(N)-VT(N1))/VT(N)

CALCULATE THE DEPENDENT VARIABLE ON THE SUB-INTERVAL LEVEL
DO 50 I=1,N
VT(I1)=VT(I1)*DEXP(-TS*M(I1))

SUM THE DRIVE NUMBERS
DO 60 I=1,N
P(I)=P(I)+M(I)

END
CONTINUE

CALCULATE THE DEPENDENT VARIABLE ON THE NEXT COMPLETE TIME STEP

DO 70 I=1,N
V(I) = V(I) * DEXP(-TS*P(I))
CONTINUE
ITMAX = ITMAX + 1
IF (ITMAX .LT. IPR) GO TO 20

PRINT THE RESULTS

DO 71 I=1,N
THE(I) = V(I)
ITMAX = 0
WRITE(6,5)
FORMAT(/)
5 FORMAT(1X,'TIME STEP NUMBER=',I3)
WRITE(6,31) L
FORMAT(1X,'TIME = ',F10.4)
WRITE(6,32) TIME
FORMAT(1X,'ELAPSED TIME=',F10.4,' SECONDS')
32 FORMAT(1X,'TIME STEP NUMBER=',I3)
WRITE(6,31) L
FORMAT(1X,'TIME = ',F10.4)
WRITE(6,32) TIME
FORMAT(1X,'ELAPSED TIME=',F10.4,' SECONDS')

IF (N.GT.11) N21 = N/2
IF (N.GT.11) GO TO 81
WRITE(6,82)(THE(I), I=1,N)
FORMAT(11(2X,F8.6))
GO TO 84
81 CONTINUE
WRITE(6,82)(THE(I), I=1,N21)
NNS = N21 + 1
WRITE(6,82)(THE(I), I=NNS,N)
84 CONTINUE
20 CONTINUE
RETURN
END
SOURCE.EFD2D
WRITTEN BY R.F. HANDSCHUH

***** PROGRAM #2 *****

THIS PROGRAM IS FOR 2-DIMENSIONAL CARTESIAN COORDINATES
UNSTEADY STATE HEAT TRANSFER. THE METHOD OF SOLUTION IS THE
EXPONENTIAL FINITE DIFFERENCE ALGORITHM. THIS PARTICULAR PROGRAM IS
FOR INFINITE HEAT TRANSFER COEFFICIENT AT THE EXPOSED SURFACES
AT X=Y=1.0 FOR X=Y=0 THE SURFACE IS CONSIDERED TO BE
PERFECTLY INSULATED.

IMPLICIT REAL*8(A-H,O-Z)
REAL*8 V(25,25)

INPUT PROGRAM DATA

WRITE(6,15)
15 FORMAT(IX,'NUMBER OF NODES=N     I3')
READ(9,15)N
WRITE(6,12)
12 FORMAT(IX,'NUMBER OF TIME SUB INTERVALS= NS     I3')
READ(9,12)NS
WRITE(6,16)
16 FORMAT(IX,'TOTAL NUMBER OF TIME STEPS= NTOT     I3')
READ(9,16)NTOT
WRITE(6,24)
24 FORMAT(IX,'INPUT TIME*TERMAL DIFFUSIVITY / LENGTH SQUARED     F5.3')
READ(9,24)TSI
WRITE(6,22)
22 FORMAT(IX,'TOTAL TIME OF ONE TIME STEP= T     F6.4')
READ(9,22)T
WRITE(6,26)
26 FORMAT(IX,'NUMBER OF TIME STEPS BEFORE PRINTING= I3')
REAL(9,26)TPR
WRITE(6,25)
25 FORMAT(IX,'# OF NODES=',I3,2X,'# OF SUB-TIME-INT=',I3,2X,
*'# OF TIME STEPS=',I3)
WRITE(6,25)N,NS,NTOT
WRITE(6,25)TSI,T
WRITE(6,25)N1=N-1

INITIALIZE THE BOUNDARY CONDITIONS
SUBROUTINE EFDIHC

IMPLICIT REAL*8(A-H,O-Z)


CALL EXP FIN DIF FOR INFINITE HEAT TRANSFER COEFFICIENT

CALL EFDIHC(N,NS,NTOT,TSI,V,T,IPR)
STOP
END

SUBROUTINE EFDIHC(N,NS,NTOT,TSI,V,T,IPR)

WRITE(6,222) TS=TSI/DFLOAT(NS+1)
NS=NS+1

BEGIN MAIN TIME STEP LOOP

DO 20 I=1,NTOT

ZERO THE DRIVE NUMBERS AND SET TEMPORARY DEPENDENT VARIABLE EQUAL TO THE LAST FULL TIME STEP VALUE

DO 15 J=1,N
DO 15 I=1,N
P(I,J)=0.
DO 10 J=1,N
DO 10 I=1,N
VT(I,J)=V(I,J)

SUB TIME INTERVAL

DO 30 K=1,NS1

CALCULATE THE DRIVE NUMBERS
DO 41 J=2,N1
JM1=J-1
JP1=J+1
DO 40 I=2,N1
IM1=I-1
IP1=I+1
M(I,J)=(VT(IP1,J)+VT(IM1,J)+VT(I,JP1)+VT(I,JM1)-4.*VT(I,J))
M(I,J)=M(I,J)/VT(I,J)
40 CONTINUE

C INSULATED BOUNDARY ALONG X-AXIS
C
J=1
JP1=J+1
DO 42 I=2,N1
IP1=I+1
IM1=I-1
M(I,J)=(VT(IP1,J)+VT(IM1,J)+2.*VT(I,JP1)-4.*VT(I,J))/VT(I,J)
42 CONTINUE
C
C INSULATED BOUNDARY ALONG Y-AXIS
C
I=1
IP1=I+1
DO 43 J=2,N1
JP1=J+1
JM1=J-1
M(I,J)=(2.*VT(IP1,J)+VT(I,JP1)+VT(I,JM1)-4.*VT(I,J))/VT(I,J)
43 CONTINUE
C
C CORNER AT ORIGIN
C
M(1,1)=(2.*VT(1,2)+2.*VT(2,1)-4.*VT(1,1))/VT(1,1)
C
C CALCULATE THE DEPENDENT VARIABLE ON THE SUB-INTERVAL LEVEL
C
DO 50 I1=1,N1
DO 50 J1=1,N1
VT(I1,J1)=VT(I1,J1)*DEXP(TS*M(I1,J1))
50 CONTINUE
C
C SUM THE DRIVE NUMBERS
C
DO 60 I=1,N1
DO 60 J=1,N1
P(I,J)=P(I,J)+M(I,J)
60 CONTINUE
C
C CALCULATE THE DEPENDENT VARIABLE AT THE NEXT COMPLETE TIME STEP
C
DO 70 J=1,N
DO 70 I=1,N
V(I,J)=V(I,J)*DEXP(TS*P(I,J))
70 CONTINUE
ITMAX=ITMAX+1
IF(ITMAX.LT.IPR)GO TO 20
C PRINT THE RESULTS
C
5 WRITE(6,5)
   FORMAT(/)
31 WRITE(6,31)L
   FORMAT(1X,'TIME STEP NUMBER=',I3/)
   TIME=T*DFLOAT(L)
   WRITE(6,32)TIME
32 WRITE(5X,'ELAPSED TIME=',F10.4,'SECONDS'/)
   DO 71 I=1,N
   DO 71 J=1,N
71 THE(I,J)=1.0-V(I,J)
   IF(N.GT.11)GO TO 58
   DO 59 J=1,N
   WRITE(6,82)(THE(I,J),I=1,N)
59 CONTINUE
   GO TO 54
58 CONTINUE
   DO 57 J=1,N
   WRITE(6,56)(THE(I,J),I=1,N)
56 CONTINUE
   GO TO 54
54 CONTINUE
20 ITMAX=0
   CONTINUE
   RETURN
END
SOURCE.LAPLAC

WRITTEN BY R.F. HANDSCHUH

***** PROGRAM #3 *****

THIS PROGRAM IS TO BE USED TO SOLVE THE LAPLACE'S EQUATION USING THE EXPONENTIAL FINITE DIFFERENCE METHOD.

IMPLICIT REAL*8(A-H,O-Z)
REAL*8 V(25,25)

INPUT PROGRAM DATA

WRITE(6,15)
15 FORMAT(1X,'NUMBER OF NODES= N I3'/)
READ(9,10)N
10 FORMAT(I3)
WRITE(6,12)
12 FORMAT(1X,'NUMBER OF TIME SUB INTERVALS= NS I3')
READ(9,13)NS
13 FORMAT(I3)
WRITE(6,16)
16 FORMAT(1X,'TOTAL NUMBER OF TIME STEPS= NTOT I3')
READ(9,21)NTOT
21 FORMAT(I3)
WRITE(6,24)
24 FORMAT(1X,'INPUT TIME / LENGTH SQUARED F5.3')
READ(9,25)TSI
25 FORMAT(F5.3)
WRITE(6,22)
22 FORMAT(1X,'TOTAL TIME OF ONE TIME STEP= T F7.6')
READ(9,23)T
23 FORMAT(F7.6)
WRITE(6,26)
26 FORMAT(1X,'NUMBER OF TIME STEPS BEFORE PRINTING= I3')
READ(9,27)IPR
27 FORMAT(I3)
WRITE(6,31)
31 FORMAT(1X,'INPUT ACCURACY DESIRED= F7.6')
READ(9,32)ET1
32 FORMAT(F7.6)
N1=N-1

INITIALIZE THE BOUNDARY CONDITIONS

DO 30 I=2,N1
DO 30 J=2,N1
30 V(I,J)=100.

DO 50 J=1,N
V(I,J)=0.0
J=N
DO 51 I=1,N
V(I,J)=0.0

J=1
DO 53 I=2,N1
V(I,J)=0.0

I=1
DO 52 J=1,N
V(I,J)=100.0

CALL EXP FIN DIF FOR LAPLACE EQUATION
CALL EFDIHC(N,NS,NTOT,TSI,V,T,IPR,ETL)
STOP
END

SUBROUTINE EFDIHC(N,NS,NTOT,TSI,V,T,IPR,ETL)
IMPLICIT REAL*8(A-H,O-Z)
TS=TS1/DFLOATCNS+1)
N1=N-1
NS1=NS+1

BEGIN MAIN STEP INCREMENT
DO 20 L=1,NTOT
ETA=0.0

ZERO THE SUM OF THE DRIVE NUMBERS
DO 15 J=1,N
DO 15 I=1,N
P(I,J)=0.

SAVE THE DEPENDENT VALUES FROM THE LAST TOTAL TIME STEP
DO 10 J=1,N
DO 10 I=1,N
DELV=V(I,J)-VT(I,J)
ETA=ETA+DABS(DELV)
10 VT(I,J)=V(I,J)
IF(MAXL.LE.88) GO TO 107
IF(ETA.LE.ETL)GO TO 100
CONTINUE
SUB TIME INTERVAL
DO 30 K=1,NS1
CALCULATE THE DRIVE NUMBERS
DO 41 J=2,N1
JM1=J-1
JP1=J+1
DO 40 I=2,N1
IM1=I-1
IP1=I+1
M(I,J)=(VT(IP1,J)+VT(IM1,J)+VT(I,JP1)+VT(I,JM1)-4.*VT(I,J))
40 M(I,J)=M(I,J)/VT(I,J)
41 CONTINUE
CALCULATE THE DEPENDENT VARIABLE ON THE SUB-TIME INTERVAL
DO 50 J1=2,N1
50 V(I1,J1)=V(I1,J1)*DEXPCTS*M(I1,J1)
SUM THE DRIVE NUMBERS
DO 60 I=1,N1
DO 60 J=1,N1
60 PCI,J)=P(I,J)+M(I,J)
30 CONTINUE
CALCULATE THE DEPENDENT VARIABLE ON THE NEXT TOTAL STEP
DO 70 J=1,N
DO 70 I=1,N
70 THE(I,J)=V(I,J)*DEXPCTS*P(I,J)
CONTINUE
ITMAX=ITMAX+1
PRINT THE RESULTS
IF(ITMAX.LT.IPR)GO TO 20
WRITE(6,5)
5 FORMAT(/)
WRITE(6,31)L
31 FORMAT(1X,'TIME STEP NUMBER=',I3/) TIME=T*DFLOAT(L)
DO 71 I=1,N
DO 71 J=1,N
71 THE(I,J)=V(I,J)
IF(N.GT.11)GO TO 58
DO 59 J=1,N
WRITE(6,82)(THE(I,J),I=1,N)
58 FORMAT(11(2X,F8.4))
59 CONTINUE
GO TO 54
CONTINUE
DO 57 J=1,N
WRITE(6,56)(THE(I,J),I=1,11)
56 FORMAT(11(2X,F8.4))
WRITE(6,56)(THE(I,J),I=12,N)
57 CONTINUE
54 CONTINUE
ITMAX=0
20 CONTINUE
GO TO 101
100 CONTINUE
WRITE(6,103)L
103 FORMAT(2X,'**** CONVERGED RESULT *****',5X,'ITERATION=',I4/)
DO 102 J=1,N
WRITE(6,56)(V(I,J),I=1,N)
102 CONTINUE
101 CONTINUE
RETURN
END
WRITTEN BY R.F. HANDSCHUH

***** PROGRAM #4 *****

SOURCE.EFDVAR

THIS PROGRAM IS FOR THE SOLUTION OF THE DIFFUSION EQUATION
WITH VARYING THERMAL CONDUCTIVITY. THE METHOD USED IS THE
EXPONENTIAL FINITE DIFFERENCE METHOD.

IMPLICIT REAL*8(A-H,O-Z)
REAL*8 V(100),KR

INPUT PROGRAM DATA

WRITE(6,15) FORMAT(1X,'NUMBER OF NODES=N I3')
READ(9,10)N
WRITE(6,12) FORMAT(I3)
10 FORMAT(1X,'NUMBER OF TIME SUB INTERVALS= NS I3')
READ(6,13)NS
WRITE(6,16) FORMAT(I3)
13 FORMAT(1X,'TOTAL NUMBER OF TIME STEPS= NTOT I3')
READ(9,21)NTOT
WRITE(6,24) FORMAT(1X,'INPUT TIME*THERMAL DIFFUSIVITY / LENGTH SquARED F5.3')
READ(9,25)TSI
WRITE(6,22) FORMAT(I3)
24 FORMAT(1X,'TOTAL TIME OF ONE TIME STEP= T F7.5')
READ(6,23)T
WRITE(6,26) FORMAT(1X,'INPUT THERMAL CONDUCTIVITY SLOPE VALUE= F5.4')
READ(9,27)BETA
WRITE(6,14) FORMAT(F5.3)
14 FORMAT(1X,'INPUT REFERENCE THERMAL CONDUCTIVITY= F5.4')
READ(9,17)KR
WRITE(6,27) FORMAT(F5.4)
17 FORMAT(F5.4)
WRITE(6,11) FORMAT(F5.4)

INITIALIZE THE BOUNDARY CONDITIONS

V(1)=0.
V(N)=0.
N1=N-1
DO 30 I=2,N1
30 V(I)=100.
CALL EXP FIN DIF FOR INFINITE HEAT TRANSFER COEFFICIENT

CALL EFDIHC(N,NS,NTOT,TSI,V,T,KR,BETA)
STOP
END

SUBROUTINE EFDIHC
SUBROUTINE EFDIHC(N,NS,NTOT,TSI,V,T,KR,BETA)

FOR INFINITE HEAT TRANSFER COEFFICIENT

IMPLICIT REAL*8(A-H,O-Z)
REAL*8 V(100),M(100),P(100)
REAL*8 KR,K(100),K0(100),THE(100),KAPPA(100)
TS=TSI/DFLOAT(NS+1)
N1=N-1
NS1=NS+1

PRINT HEADING

WRITE(6,5)
WRITE(6,100)
FORMAT(1X,'***** SOURCE.EFDVAR *****')

BEGIN MAIN TIME STEP LOOP

DO 20 L=1,NTOT

ZERO THE SUM OF THE DRIVE NUMBERS

DO 15 I=1,N
P(I)=0.

SET VARIABLES EQUAL TO THE LAST STEPS VALUES

DO 10 I=1,N
VT(I)=V(I)
DO 11 I=1,N
K0(I)=KR+BETA*KR*V(I)

SUB TIME INTERVAL

DO 30 KK=1,NS1
DO 35 I=1,N
K(I)=KR+BETA*KR*VT(I)
THE(I)=VT(I)+BETA*VT(I)*VT(I)/2.0

CALCULATE THE DRIVE NUMBERS

DO 40 I=2,N1
IM1=I-1
IP1=I+1
M(I) = VT(I+1) + VT(I-1) - 2 * VT(I) / 2.
M(I) = M(I) + BETA * (VT(I+1) + VT(I-1) - 2 * VT(I)) / 2.
M(I) = M(I) / THE(I)
DO 50 I = 2, N
50 KAPPA(I) = THE(I) * DEXP(TS * (1 + BETA * VT(I)) * M(I))

CALCULATE THE DEPENDENT VARIABLE ON THE SUB-TIME INTERVAL

DO 55 I = 2, N
55 VT(I) = (-1 + SQRT(1 + 2 * KAPPA(I) * BETA)) / BETA

SUM THE DRIVE NUMBERS
DO 60 I = 2, N
60 P(I) = P(I) + M(I)
CONTINUE
WRITE(6, 5)
5 FORMAT(/)
WRITE(6, 31) L
31 FORMAT(1X, 'TIME STEP NUMBER=', I3/)
TIME = T * DFLOAT(L)
WRITE(6, 32) TIME
32 FORMAT(5X, 'ELAPSED TIME=', F10.4, ' SECONDS'/)

CALCULATE THE NEXT TOTAL STEP DEPENDENT VARIABLES AND PRINT RESULTS
DO 70 I = 1, N
70 THE(I) = V(I) + BETA * V(I) * V(I) / 2.0
KAPPA(I) = THE(I) * DEXP(TS * (1 + BETA * V(I)) * P(I))
V(I) = (-1 + SQRT(1 + 2 * KAPPA(I) * BETA)) / BETA
WRITE(6, 81)(V(I), I = 1, N)
81 FORMAT(1X, 11(F8.5, 2X))
CONTINUE
RETURN
END
WRITTEN BY R.F. HANDSCHUH

SOURCE.EXPVAR

***** PROGRAM #5 *****

THIS PROGRAM IS FOR THE SOLUTION OF THE DIFFUSION EQUATION WITH VARYING THERMAL CONDUCTIVITY. THE METHOD USED IS THE EXPLICIT FINITE DIFFERENCE METHOD.

IMPLICIT REAL*8(A-H,O-Z)
REAL*8 VOOO),KR

INPUT PROGRAM DATA

WRITE(6,15)
15 FORMAT(1X,'NUMBER OF NODES=N I3'/)
READ(9,10)N
10 FORMAT(I3)
WRITE(6,16)
16 FORMAT(1X,'TOTAL NUMBER OF TIME STEPS= NTOT I3'/)
READ(9,21)NTOT
21 FORMAT(I3)
WRITE(6,24)
24 FORMAT(1X,'INPUT TIME* THERMAL DIFFUSIVITY / LENGTH SQUARED F5.3'/)
READ(9,25)TSI
25 FORMAT(F5.3)
WRITE(6,22)
22 FORMAT(1X,'TOTAL TIME OF ONE TIME STEP= T F7.5'/)
READ(6,23)T
23 FORMAT(F7.5)
WRITE(6,14)
14 FORMAT(1X,'INPUT REFERENCE THERMAL CONDUCTIVITY=F5.4'/)
READ(9,17)KR
17 FORMAT(F5.4)
WRITE(6,26)
26 FORMAT(1X,'INPUT THERMAL CONDUCTIVITY SLOPE VALUE= F5.4'/)
READ(9,27)BETA
27 FORMAT(F5.4)

INITIALIZE THE BOUNDARY CONDITIONS

V(1)=0.
V(N)=0.
N1=N-1
DO 30 I=2,N1
30 V(I)=100.

CALL EXP FIN DIF FOR INFINITE HEAT TRANSFER COEFFICIENT
CALL EFDIHC(N, NTOT, TSI, V, T, KR, BETA)
STOP
END

SUBROUTINE EFDIHC
SUBROUTINE EFDIHC(N, NTOT, TSI, V, T, KR, BETA)

FOR INFINITE HEAT TRANSFER COEFFICIENT

IMPLICIT REAL*8(A-H, O-Z)
REAL*8 VT(100), VC100), M(100), P(100)
REAL*8 KR
N1=N-1

BEGIN TIME STEP LOOP
DO 20 L=1, NTOT

CALCULATE DEPENDENT VARIABLE
DO 40 I=2, N1
IM1=I-1
IP1=I+1
VT(I)=V(I)+TSI*((1.0+BETA*V(I))*(V(IP1)+V(IM1))-2.0*V(I))
DERSQR=DABS(V(IP1)-V(IM1))
IF(DERSQR.LE.0.0)GO TO 40
VT(I)=VT(I)+TSI*((BETA*(DERSQR)**2.0)/4.)
CONTINUE

PRINT RESULTS
WRITE(6, 5)
FORMAT(/)
WRITE(6, 31)L
FORMAT(1X, 'TIME STEP NUMBER=', I3/)
WRITE(6, 32)L
FORMAT(5X, 'ELAPSED TIME=', F10.4, 'SECONDS'/)
DO 70 I=1, N
70 V(I)=VT(I)
WRITE(6, 81)(V(I), I=1, N)
FORMAT(1X, 11(F9.5, 2X))
CONTINUE
RETURN
END
SOURCE. IMPVAR

WRITTEN BY R.F. HANDSCHUH

***** PROGRAM #6 *****

THIS ROUTINE IS TO BE USED FOR COMPARISION TO EXPONENTIAL
FINITE DIFFERENCE ALGORITHM. THIS ROUTINE WILL USE THE
IMPLICIT ROUTINE TO SOLVE FOR THE TEMPERATURE FIELD USING
THE TRI-DIAGONAL MATRIX ALGORITHM.

IMPLICIT REAL*8(A-H,O-Z)
REAL*8 V(100),KR

INPUT PROGRAM DATA

WRITE(6,15)
15 FORMAT(1X,'NUMBER OF NODES=N  I3'/)
10 FORMAT(I3)
WRITE(6,16)
16 FORMAT(1X,'TOTAL NUMBER OF TIME STEPS= NTOT  I3')
21 FORMAT(I3)
WRITE(6,24)
24 FORMAT(1X,'INPUT TIME*THERMAL DIFFUSIVITY / LENGTH SQUARED  F5.3')
25 FORMAT(F5.3)
WRITE(6,22)
22 FORMAT(1X,'TOTAL TIME OF ONE TIME STEP= T F7.5')
23 FORMAT(F7.5)
WRITE(6,14)
14 FORMAT(1X,'INPUT REFERENCE THERMAL CONDUCTIVITY=F5.4')
17 FORMAT(F5.4)
WRITE(6,26)
26 FORMAT(1X,'INPUT THERMAL CONDUCTIVITY SLOPE VALUE= F5.4')
27 FORMAT(F5.4)

INITIALIZE BOUNDARY CONDITIONS

V(1)=0.
V(N)=0.
M1=N-1
DO 30 I=2,M1
30 V(I)=100.

CALL IMPLICIT ROUTINE
CALL IMPL(N, NTOT, TSI, V, TS, KR, BETA)
STOP
END

SUBROUTINE IMPL
SUBROUTINE IMPL(N, NTOT, OMEGA, T, TS, KR, BETA)

FOR INFINITE HEAT TRANSFER COEFFICIENT AND VARYING THERMAL CONDUCTIVITY

IMPLICIT REAL*8(A-H, O-Z)
REAL*8 A(101), B(101), C(101), D(101), KAP(101), GAM(101), T(101)
REAL*8 KR

PRINT HEADING
WRITE(6, 331)
FORMAT(1X, '***** SOURCE.IMPVAR *****'
RHO=1.0
CP=1.0
N1=N-1

BEGIN TIME STEP LOOP
DO 20 L=1, NTOT
CALCULATE THOMAS ALGORITHM VARIABLES AND THOSE THAT ARE A FUNCTION OF TEMPERATURE

DO 21 I=1, N
D(I)=T(I)
DO 200 I=1, N
TO(I)=T(I)
DO 25 I=2, N1
KAP(I)=1.0+BETA*T(I)
GAM(I)=BETA*(T(I+1)-T(I-1))/4.0

DO 30 I=2, N1
A(I)=-OMEGA*(KAP(I)+GAM(I))
B(I)=(1.0+BETA*KAP(I))
C(I)=-OMEGA*(KAP(I)+GAM(I))

CALL TRI-DIAGONAL-MATRIX ALGORITHM
CALL TRIDAG(2, N1, A, B, C, D, T)
CONTINUE

PRINT THE RESULTS
WRITE(6, 5)
FORMAT(*/

-- -- -- -- --
SUBROUTINE TRIDAG

THIS ROUTINE IS FOR THE SOLUTION OF THE THOMAS ALGORITHM

THIS ROUTINE WAS TAKEN FROM THE BOOK APPLIED NUMERICAL METHODS
BY CARNAHAN, LUTHER, AND WILKES.

SUBROUTINE TRIDAG(IF,L,A,B,C,D,V)

IMPLICIT REAL*8(A-H,O-Z)

REAL*8 A(101),B(101),C(101),D(101),V(101),BETA(101),GAMMA(101)

COMPUTE INTERMEDIATE ARRAYS BETA AND GAMMA

BETA(IF)=B(IF)
GAMMA(IF)=D(IF)/BETA(IF)
IFP1=IF+1
DO 1 I=IFP1,L
BETA(I)=B(I)-A(I)*C(I-1)/BETA(I-1)
1 GAMMA(I)=(D(I)-A(I)*GAMMA(I-1))/BETA(I)

COMPUTE FINAL SOLUTION VECTOR V

V(L)=GAMMA(L)
LAST=L-IF
DO 2 K=1,LAST
I=L-K
V(I)=GAMMA(I)-C(I)*V(I+1)/BETA(I)
2 RETURN
END
SOURCE.COU.

WRITTEN BY R.F. HANDSCHUH

***** PROGRAM #7 *****

THIS PROGRAM IS TO BE USED TO DEMONSTRATE THE USE OF THE
EXPONENTIAL FINITE DIFFERENCE METHOD ON THE DEVELOPING TEMPERATURE
FIELD IN A LAMINAR COUETTE FLOW

IMPLICIT REAL*8(A-H,O-Z)
REAL*8 V(100)

INPUT PROGRAM DATA

WRITE(6,15)
15 FORMAT(1X,'NUMBER OF NODES=N I3')
READ(9,10)N
10 FORMAT(I3)
WRITE(6,12)
12 FORMAT(1X,'NUMBER OF SUB INTERVALS= NS I3')
READ(6,13)NS
13 FORMAT(I3)
WRITE(6,16)
16 FORMAT(1X,'TOTAL NUMBER OF POSITION STEPS= NTOT I3')
READ(9,21)NTOT
21 FORMAT(I3)
WRITE(6,24)
24 FORMAT(1X,'INPUT (POSITION STEP*KIN. VISC)/(LENGTH SQUARED) F5.3')
READ(9,25)TSI
25 FORMAT(F5.3)
WRITE(6,22)
22 FORMAT(1X,'TOTAL DISTANCE OF ONE STEP= T F5.3')
READ(6,23)T
23 FORMAT(F5.3)
WRITE(6,14)
14 FORMAT(1X,'INPUT (MAX WIDTH)/(FLOW VEL) F5.4')
READ(6,17)SP
17 FORMAT(F5.4)
WRITE(6,31)
31 FORMAT(1X,'INPUT INTERVAL FOR PRINTING RESULTS= I3')
READ(6,32)IPR
32 FORMAT(I3)

INITIALIZE THE BOUNDARY CONDITIONS

V(1)=0.
DO 30 I=2,N
30 V(I)=100.
CALL EXPONENTIAL FINITE DIFFERENCE FOR COUETTE FLOW
CALL EFFL(N,NS,NTOT,TSI,V,T,SP,IPR)
STOP
END

SUBROUTINE EFFL
SUBROUTINE EFFL(N,NS,NTOT,TSI,V,T,SP,IPR)
FOR COUETTE LAMINAR FLOW
IMPLICIT REAL*8(A-H,O-Z)
REAL*8 VT(100),V(100),M(100),P(100),FL(20)
TS=TSI/DFLOAT(NS+1)
N1=N-1
PRINT HEADING
WRITE(6,92)
FORMAT(1X,'SOLUTION FOR DEVELOPING TEMPERATURE FIELD IN'\,*'LAMINAR COUETTE FLOW'/)//
DY=1./DFLOAT(N-1)
CALCULATE PARAMETER THAT VARIES WITH POSITION IN THE FLOW
DO 115 I=2,N
115 FL(I)=SP/(DY*DFLOAT(I-1))
NS1=NS+1
BEGIN TOTAL POSITION STEP LOOP
DO 20 L=1,NTOT
IT=IT+1
ZERO THE SUM OF DRIVE NUMBERS
DO 15 I=1,N
P(I)=0.
15 SET TEMPORARY VALUES EQUAL TO THE LAST POSITION STEP VALUE
DO 10 I=1,N
VT(I)=V(I)
10 SUB POSITION INTERVAL
DO 30 K=1,NS1
CALCULATE THE DRIVE NUMBERS
DO 40 I=2,N1
IM1=I-1
IP1=I+1
M(I)=(2.*VT(I)-VT(IM1)-VT(IP1))/VT(I)
C
C CALCULATE TEMPORARY DEPENDENT VARIABLE ON SUB-INTERVAL
C
DO 50 I1=2,N1
VT(I1)=VT(I1)*DEXP(-TS*M(I1)*FL(I1))
DO 60 I=2,N1
P(I)=P(I)+M(I)
CONTINUE
C
C CALCULATE THE COMPLETE STEP DEPENDENT VARIABLES
C
DO 70 I=1,N
V(I)=V(I)*DEXP(-TS*P(I)*FL(I))
CONTINUE
IF(IT.LT.IPR)GO TO 20
IT=0
C
C PRINT THE RESULTS
C
WRITE(6,5)
FORMAT(/)
WRITE(6,31)L
FORMAT(1X,'POSITION STEP NUMBER=',I3)
DIST=T*DFLOAT(L)
WRITE(6,32)DIST
FORMAT(1X,'LOCATION DIST=',F10.4,'METERS')
WRITE(6,82)(V(I),I=1,N)
CONTINUE
RETURN
END
WRITTEN BY R. F. HANDSCHUH
SOURCE.EX3D

***** PROGRAM #8 *****

THIS ROUTINE IS FOR FINDING THE TEMPERATURE AT A GIVEN LOCATION
AND TIME FOR A 3-DIMENSIONAL SOLID.

THIS PROGRAM CALCULATES THE EXACT TEMP AS FOUND IN THE BOOK
"TRANSPORT PHENOMENA" BY BIRD, STEWART, AND LIGHTFOOT.

IMPLICIT REAL*8(A-H,O-Z)
REAL*8 PI,PI2,PI3
PI=3.1415926

INPUT PROGRAM DATA

31 FORMAT(1X,'NUMBER OF NODES PER COORDINATE DIRECTION=I3')
32 FORMAT(I3)

1 FORMAT(1X,'INPUT INITIAL TEMPERATURE FOR THE SOLID F6.3')
2 FORMAT(F6.3)

3 FORMAT(1X,'SURFACE TEMPERATURE FOR TIME > 0 SECONDS F6.3')
4 FORMAT(F6.3)

5 FORMAT(1X,'INPUT THERMAL DIFFUSIVITY F5.3')
6 FORMAT(F5.3)

ASSUMING 3 DIM. CUBE

DATA A,B,C/1.0,1.0,1.0/
WRITE(1,9)
9 FORMAT(1X,'INPUT THE NUMBER OF TIMES THROUGH SUMMATION I3')
11 FORMAT(I3)

12 FORMAT(1X,'INPUT TIME TO BE EVALUATED AT F5.3')
14 FORMAT(F5.3)

20 FORMAT(1X,'TEMP INITIAL=',E15.8,2X,'TEMP SURF=',E15.8/

30 FORMAT(1X,'THER DIFF=',E15.8,2X,'TIME=',F7.5)
START SUMMATION ROUTINE

PI3=PI**3.
PI2=PI**2.

CALCULATE THE TEMPERATURE ALONG THE DIAGONAL

DEL=A/(DFLOAT(NODES-1))

DO 100 NODE=1,NODES
   X=DEL*DFLOAT(NODE-1)
   Y=X
   Z=X
   SUM=0.0
   DO 10 K=1,N1
      RP=DFLOAT(K-1)
      RPI=RP+.5
      COSP=DCOS(RPI*PI*Z/C)
   DO 10 M=1,N1
      RN=DFLOAT(N-1)
      RN1=RN+.5
      COSN=DCOS(RN1*PI*Y/B)
   DO 10 11=1,111
      RM=DFLOAT(M-1)
      RM1=RM+.5
      COSM=DCOS(RM1*PI*X/A)
   J=M+N+K-3
   J1=J/2
   J2=J1*2
   VAL=-1.0
   IF(J.EQ.J2)VAL=1.0
   GAM=(RM1**2)/(A*A)+(RN1**2)/(B*B)+(RPI**2)/(C*C)
   EXPLIM=(-GAM*PI2*ALFA*TIME)
   IF(EXPLIM.LT.-100.)GO TO 999
   EP=DEXP(EXPLIM)
   GO TO 998
999 CONTINUE
   EP=0.0
998 CONTINUE
   SUM=SUM+(VAL/(RM1*RN1*RPI*PI3))*EP*COSM*COSN*COSP
10 CONTINUE
   TNEW=T1+8.*(T0-T1)*SUM

PRINT THE RESULTS

WRITE(6,16)X,Y,Z
16 FORMAT(1X,'X=',F5.3,2X,'Y=',F5.3,2X,'Z=',F5.3)
   IF(TNEW.LT.1E-10)TNEW=0.0
   WRITE(6,15)TNEW
15 FORMAT(1X,'TEMPERATURE=',E15.8)
100 CONTINUE
STOP
END
PROCEDURE FOR 3-DIMENSIONAL CARTESIAN COORDINATES
UNSTABLE STATE HEAT TRANSFER IN A CUBE. THE METHOD OF SOLUTION IS THE
EXPOENTIAL FINITE DIFFERENCE ALGORITHM. THIS PARTICULAR PROGRAM IS
FOR INFINITE HEAT TRANSFER COEFFICIENT AT THE EXPOSED SURFACES
AT X=Y=Z=1.0 FOR THE THREE SURFACES WHERE X,Y,Z EQUAL 0.0
ARE TO BE CONSIDERED AS PERFECTLY INSULATED.

IMPLICIT REAL*8(A-H,O-Z)
REAL*8 V(25,25,25)

INPUT PROGRAM DATA

WRITE(6,15)
15 FORMAT(1X,"NUMBER OF NODES=N I3")
READ(9,10)N
WRITE(6,12)
12 FORMAT(1X,"NUMBER OF TIME SUB INTERVALS= NS I3")
READ(9,13)NS
WRITE(6,16)
16 FORMAT(1X,"TOTAL NUMBER OF TIME STEPS= NTOT I3")
READ(9,21)NTOT
WRITE(6,24)
24 FORMAT(1X,"INPUT TIME*THERMAL DIFFUSIVITY \ LENGTH SQUARED F5.3")
READ(9,25)TSI
WRITE(6,22)
22 FORMAT(1X,"TOTAL TIME OF ONE TIME STEP= T F6.4")
READ(9,23)T
WRITE(6,26)
26 FORMAT(1X,"NUMBER OF TIME STEPS BEFORE PRINTING= I3")
READ(9,27)IPR
WRITE(6,250)N,NS,NTOT
250 FORMAT(1X, '# OF NODES=',I3,2X,'# OF SUB-TIME-INT=',I3,2X,
    '# OF TIME STEPS=',I3)
WRITE(6,251)TSI,T
251 FORMAT(1X,"TIME*THERMAL DIFF/LENGTH SQUARED='F5.3,2X,
    'TIME STEP LENGTH=' ,F6.4")
N1=N-1

INITIALIZE BOUNDARY CONDITIONS
DO 30 I=1,N1
DO 30 J=1,N1
DO 30 K=1,N1
30 V(I,J,K)=1.0
C
I=N
DO 50 J=1,N
DO 50 K=1,N
50 V(I,J,K)=0.0
J=N
DO 51 I=1,N
DO 51 K=1,N
51 V(I,J,K)=0.0
K=N
DO 52 I=1,N
DO 52 J=1,N
52 V(I,J,K)=0.0
CALL EXP FIN DIF FOR INFINITE HEAT TRANSFER COEFFICIENT
CALL EFDIHC(N,NS,NTOT,TSI,V,T,IPR)
STOP
END
SUBROUTINE EFDIHC
SUBROUTINE EFDIHC(N,NS,NTOT,TSI,V,T,IPR)
FOR INFINITE HEAT TRANSFER COEFFICIENT
IMPLICIT REAL*8(A-H,O-Z)
REAL*8 M1,M2
TS=TSI/DFLOAT(NS+1)
N1=N-1
NS1=NS+1
PRINT HEADING
WRITE(6,200)
200 FORMAT(1X,'********** RESULTS FROM EFD3D **********'/)
START TOTAL TIME STEP LOOP
DO 20 L=1,NTOT
ZERO THE SUM OF THE SUB-INTERVAL DRIVE NUMBERS
DO 15 K=1,N
DO 15 J=1,N
DO 15 I=1,N
15 P(I,J,K)=0.
SET SUB-INTERVAL VALUES EQUAL TO THE LAST TIME STEP VALUES
DO 10 K=1,N
DO 10 J=1,N
DO 10 I=1,N

VT(I,J,K)=V(I,J,K)

SUB TIME INTERVAL
CALCULATE THE DRIVE NUMBERS WHICH IS DEPENDENT ON LOCATION IN THE CUBE

DO 30 KS=1,NS1
DO 42 K=2,N1
KM1=K-1
KP1=K+1
DO 41 J=2,N1
JM1=J-1
JP1=J+1
DO 40 I=2,N1
IM1=I-1
IP1=I+1
M1=VT(IP1,J,K)+VT(IM1,J,K)+VT(I,JP1,K)+VT(I,JM1,K)
M2=M1+VT(I,J,KP1)+VT(I,J,KM1)-6.*VT(I,J,K)
IF(VT(I,J,K).LE.0.0)M(I,J,K)=0.
IF(VT(I,J,K).LE.0.0)GO TO 40
M(I,J,K)=(M1-6.*VT(I,J,K))/VT(I,J,K)

CONTINUE
CONTINUE
CONTINUE

INSULATED BOUNDRY ALONG X-AXIS

J=1
K=1
KP1=K+1
JP1=J+1
DO 48 I=2,N1
IP1=I+1
IM1=I-1
M1=VT(IP1,J,K)+VT(IM1,J,K)+2.*VT(I,JP1,K)+2.*VT(I,J,KP1)
IF(VT(I,J,K).LE.0.0)M(I,J,K)=0.
IF(VT(I,J,K).LE.0.0)GO TO 48
M(I,J,K)=(M1-6.*VT(I,J,K))/VT(I,J,K)
CONTINUE

INSULATED BOUNDRY ALONG Y-AXIS

I=1
K=1
KP1=K+1
DO 43 J=2,N1
JP1=J+1
JM1=J-1
M1=2.*VT(IP1,J,K)+VT(I,JP1,K)+VT(I,JM1,K)+2.*VT(I,J,KP1)
IF(VT(I,J,K).LE.0.0)M(I,J,K)=0.
IF(VT(I,J,K).LE.0.0)GO TO 43
M(I,J,K)=(M1-6.*VT(I,J,K))/VT(I,J,K)
CONTINUE

INSULATED BOUNDARY ALONG Z-AXIS

J=1
JPL=J+1
I=1
IPL=I+1
DO 44 K=2,N1
KM=K-1
KPL=K+1
M1=2.*VT(IP1,J,K)+2.*VT(I,J,KPL)+VT(I,J,KM)
IF(VT(I,J,K).LE.0.0)M(I,J,K)=0.
IF(VT(I,J,K).LE.0.0)GO TO 44
M(I,J,K)=(M1-6.*VT(I,J,K))/VT(I,J,K)
CONTINUE

INSULATED FACE AT Z=0

K=1
KPL=K+1
DO 45 I=2,N1
IM=I-1
DO 45 J=2,N1
JM=J-1
M1=VT(IP1,J,K)+VT(IM1,J,K)+VT(I,JPL,K)+VT(I,JM1,K)+2.*VT(I,J,KPL)
IF(VT(I,J,K).LE.0.0)M(I,J,K)=0.
IF(VT(I,J,K).LE.0.0)GO TO 45
M(I,J,K)=(M1-6.*VT(I,J,K))/VT(I,J,K)
CONTINUE

AT THE FACE WHERE Y=0

J=1
JPL=J+1
DO 46 I=2,N1
IM=I-1
DO 46 K=2,N1
KM=K-1
M1=VT(IP1,J,K)+VT(IM1,J,K)+2.*VT(I,JPL,K)+VT(I,J,KPL)+VT(I,J,KM)
IF(VT(I,J,K).LE.0.0)M(I,J,K)=0.
IF(VT(I,J,K).LE.0.0)GO TO 46
M(I,J,K)=(M1-6.*VT(I,J,K))/VT(I,J,K)
CONTINUE

AT THE FACE WHERE X=0
I=1
IP1=I+1
DO 47 J=2,N1
JP1=J+1
JM1=J-1
DO 47 K=2,N1
KP1=K+1
KM1=K-1
M1=2.*VT(IP1,J,K)+VT(I,JP1,J)+VT(I,JM1,K)+VT(I,J,KP1)+VT(I,J,KM1)
IF(VT(I,J,K).LE.0.0)M(I,J,K)=0.
IF(VT(I,J,K).LE.0.0)GO TO 47
M(I,J,K)=(M1-6.*VT(I,J,K))/VT(I,J,K)
CONTINUE
C CORNER AT ORIGIN
M1=2.*VT(1,2,1)+2.*VT(2,1,1)+2.*VT(1,1,2)-6.*VT(1,1,1)
M(1,1,1)=M1/VT(1,1,1)
C CALCULATE THE SUB-INTERVAL DEPENDENT VARIABLES
DO 50 I1=1,N
DO 50 J1=1,N
DO 50 K1=1,N
IF(M(I1,J1,K1).LT.-50.)VT(I1,J1,K1)=0.0
IF(M(I1,J1,K1).LT.-50)GO TO 50
VT(I1,J1,K1)=VT(I1,J1,K1)*DEXPCTS*M(I1,J1,K1)
CONTINUE
C SUM THE DRIVE NUMBERS
DO 60 I=1,N
DO 60 J=1,N
DO 60 K=1,N
CONTINUE
C CALCULATE THE NEXT COMPLETE TIME STEP DEPENDENT VARIABLES
DO 70 K=1,N
DO 70 J=1,N
DO 70 I=1,N
IF(P(I,J,K).LT.-50.)V(I,J,K)=0.0
IF(P(I,J,K).LT.-50.)GO TO 70
V(I,J,K)=V(I,J,K)*DEXPCTS*P(I,J,K)
CONTINUE
ITMAX=ITMAX+1
IF(ITMAX.LT.IPR)GO TO 20
WRITE(6,5)
FORMAT(/)
WRITE(6,31)I
FORMAT(1X,'TIME STEP NUMBER=',I3/)
TIME=T*DFLOAT(I)
WRITE(6,32)TIME
32 FORMAT(5X,'ELAPSED TIME=','F10.4,'SECONDS')/

PRINT OUT THE DIAGONAL RESULTS

WRITE(6,82)(V(I,I,I),I=1,N)
82 FORMAT(11(2X,'F8.6'))

ITMAX=0
20 CONTINUE
RETURN
END
SOURCE.EXPL3D

WRITTEN BY R.F. HANDSCHUH

***** PROGRAM #10 *****

THIS PROGRAM IS FOR 3-DIMENSIONAL CARTESIAN COORDINATES
UNSTEADY STATE HEAT TRANSFER IN A CUBE. THE METHOD OF SOLUTION IS THE
PURE EXPLICIT METHOD. THIS PARTICULAR PROGRAM IS
FOR INFINITE HEAT TRANSFER COEFFICIENT AT THE EXPOSED SURFACES
AT X=Y=Z=1.0 FOR THE THREE SURFACES WHERE X,Y,Z EQUAL 0.0
IS TO BE CONSIDERED AS PERFECTLY INSULATED.

IMPLICIT REAL*8(A-H,O-Z)
REAL*8 V(25,25,25)

INPUT PROGRAM DATA

NUMBER OF NODES = N

READ(5,10)N
10 FORMAT(I3)

TOTAL NUMBER OF TIME STEPS = NTOT

READ(5,21)NTOT
21 FORMAT(I4)

TIME*THermal DIFFUSIVITY/LENGTH SQUARED = TSI

READ(5,25)TSI
25 FORMAT(F5.3)

TOTAL TIME OF ONE TIME STEP = T

READ(5,23)T
23 FORMAT(F6.4)

NUMBER OF STEPS BEFORE PRINTING THE RESULTS = IPR

READ(5,27)IPR
27 FORMAT(I3)

WRITE(6,252)
252 FORMAT(1X,'*************** PURE EXPLICIT FINITE DIFFERENCE ' -
*"METHOD ***************''/''/
WRITE(6,250)N,NS,NTOT
250 FORMAT(1X,'# OF NODES=',I3,2X,'# OF SUB-TIME-INT=',I3,2X, -
*'# OF TIME STEPS=',I4)
WRITE(6,251)TSI,T
251 FORMAT(1X,'(TIME*HER DIFF)/LENGTH SQUARED=','F5.3,2X, -
*TIME STEP LENGTH=','F6.4/)
N1=N-1
INITIALIZE BOUNDARY CONDITIONS

DO 30 I=1,Nl
DO 30 J=1,Nl
DO 30 K=1,Nl
30 V(I,J,K)=1.0

I=N
DO 50 J=1,N
DO 50 K=1,N
50 V(I,J,K)=0.0
J=N
DO 51 I=1,N
DO 51 K=1,N
51 V(I,J,K)=0.0
K=N
DO 52 I=1,N
DO 52 J=1,N
52 V(I,J,K)=0.0

CALL PURE EXPLICIT FINITE DIFFERENCE FOR INFINITE HEAT TRANSFER COEFFICIENT

CALL PURE(N,NTOT,TSI,V,T,IPR)
STOP
END

SUBROUTINE PURE

SUBROUTINE PURE(N,NTOT,TSI,V,T,IPR)

IMPLICIT REAL*8(A-H,O-Z)
REAL*8 V(25,25,25),VT(25,25,25)
REAL*8 M1,M2
N1=N-1

START TIME STEP LOOP

DO 20 L=1,NTOT

SAVE VALUES FROM THE LAST TIME STEP

DO 39 I=1,N
DO 39 J=1,N
DO 39 K=1,N
39 VT(I,J,K)=V(I,J,K)

CALCULATE THE FIELD VARIABLE USING THE EXPLICIT FINITE DIFFERENCE METHOD

DO 42 K=2,N1
KM1=K-1
KP1 = K + 1
DO 41 J = 2, N1
  JP1 = J + 1
  DO 40 I = 2, N1
    IP1 = I + 1
    M1 = VT(IP1, J, K) + VT(IM1, J, K) + VT(I, JP1, K) + VT(I, JM1, K)
    M2 = M1 + VT(I, J, KP1) + VT(I, J, KM1) - 6. * VT(I, J, K)
    V(I, J, K) = V(I, J, K) + TSI * M2
  40 CONTINUE
  CONTINUE
41 CONTINUE
42 CONTINUE

C INSULATED BOUNDARY ALONG X-AXIS

J = 1
  K = 1
  KP1 = K + 1
  JP1 = J + 1
  DO 48 I = 2, N1
    IP1 = I + 1
    IM1 = I - 1
    M1 = VT(IP1, J, K) + VT(IM1, J, K) + 2. * VT(I, JP1, K) + 2. * VT(I, J, KP1)
    M2 = M1 - 6. * VT(I, J, K)
    V(I, J, K) = V(I, J, K) + TSI * M2
  48 CONTINUE

C INSULATED BOUNDARY ALONG Y-AXIS

I = 1
  K = 1
  KP1 = K + 1
  JP1 = J + 1
  DO 43 J = 2, N1
    JP1 = J + 1
    JM1 = J - 1
    M1 = 2. * VT(IP1, J, K) + VT(I, JP1, K) + VT(I, JM1, K) + 2. * VT(I, J, KP1)
    M2 = M1 - 6. * VT(I, J, K)
    V(I, J, K) = V(I, J, K) + TSI * M2
  43 CONTINUE

C INSULATED BOUNDARY ALONG Z-AXIS

J = 1
  K = 1
  I = 1
  KP1 = K + 1
  DO 44 K = 2, N1
    IP1 = I + 1
    KP1 = K - 1
    M1 = 2. * VT(IP1, J, K) + 2. * VT(I, JP1, K) + VT(I, J, KP1) + VT(I, J, KM1)
    M2 = M1 - 6. * VT(I, J, K)
    V(I, J, K) = V(I, J, K) + TSI * M2
  44 CONTINUE
CONTINUE

INSULATED FACE AT Z=0

K=1
KP1=K+1
DO 45 I=2,N1
IM1=I-1
DO 45 J=2,N1
JP1=J+1
JM1=J-1
M1=VT(IP1,J,K)+VT(IM1,J,K)+VT(I,JP1,K)+VT(I,JM1,K)+2.*VT(I,J,KP1)
M2=M1-6.*VT(I,J,K)
V(I,J,K)=V(I,J,K)+TSI*M2
CONTINUE

AT THE FACE WHERE Y=0

J=1
JP1=J+1
DO 46 I=2,N1
IM1=I-1
DO 46 K=2,N1
KP1=K+1
KM1=K-1
M1=VT(IP1,J,K)+VT(IM1,J,K)+2.*VT(I,JP1,K)+VT(I,KP1)+VT(I,J,KM1)
M2=M1-6.*VT(I,J,K)
V(I,J,K)=V(I,J,K)+TSI*M2
CONTINUE

AT THE FACE WHERE X=0

I=1
IP1=I+1
DO 47 J=2,N1
JP1=J+1
JM1=J-1
DO 47 K=2,N1
KP1=K+1
KM1=K-1
M1=2.*VT(IP1,J,K)+VT(JP1,K)+VT(I,JM1,K)+VT(I,KP1)+VT(I,KM1)
M2=M1-6.*VT(I,J,K)
V(I,J,K)=V(I,J,K)+TSI*M2
CONTINUE

CORNER AT ORIGIN

M1=2.*VT(1,2,1)+2.*VT(2,1,1)+2.*VT(1,1,2)-6.*VT(1,1,1)
V(1,1,1)=V(1,1,1)+M1*TSI

ITMAX=ITMAX+1
IF(ITMAX.LT.IPR)GO TO 20
WRITE(6,5)
5 FORMAT(/)
WRITE(6,31)L
31 FORMAT(1X,'TIME STEP NUMBER=',I3/)
TIME=T*DFLOAT(L)
WRITE(6,32)TIME
32 FORMAT(5X,'ELAPSED TIME=',F10.4,'SECONDS'/)

PRINT OUT THE DIAGONAL RESULTS

WRITE(6,82)(V(I,I,I),I=1,N)
82 FORMAT(11(2X,F8.6))
ITMAX=0
20 CONTINUE
RETURN
END
SOURCE. DOUGLA

WRITTEN BY R.F. HANDSCHUH

**** PROGRAM #11 ****

THIS PROGRAM IS FOR 3-DIMENSIONAL CARTESIAN COORDINATES
UNSTEADY STATE HEAT TRANSFER IN A CUBE. THE METHOD OF SOLUTION IS THE
METHOD OF DOUGLAS. THIS PARTICULAR PROGRAM IS
FOR INFINITE HEAT TRANSFER COEFFICIENT AT THE EXPOSED SURFACES
AT X=Y=Z=1.0 FOR THE THREE SURFACES WHERE X,Y,Z EQUAL 0.0
IS TO BE CONSIDERED AS PERFECTLY INSULATED.

IMPLICIT REAL*8(A-H,O-Z)
REAL*8 T(25, 25, 25)

INPUT PROGRAM DATA

WRITE(6, 15) FORMAT(6, 15) 'NUMBER OF NODES=N I3'
READ(5, 10) N
WRITE(6, 16) FORMAT(6, 16) 'TOTAL NUMBER OF TIME STEPS= NTOT I3'
READ(5, 21) NTOT
WRITE(6, 17) FORMAT(6, 17) 'INPUT THERMAL DIFFUSIVITY / LENGTH SQUARED F5.3'
READ(5, 25) TSI
WRITE(6, 18) FORMAT(6, 18) 'TOTAL TIME OF ONE TIME STEP= T F6.4'
READ(5, 23) DT
WRITE(6, 19) FORMAT(6, 19) 'NUMBER OF TIME STEPS BEFORE PRINTING= I3'
READ(5, 27) IPR
WRITE(6, 20) FORMAT(6, 20) 'NUMBER OF NODES = ', I3, 2X, ' # OF SUB-TIME-INT= ', I3, 2X,
* " # OF TIME STEPS= ', I3) WRITE(6, 21) TSI, DT
WRITE(6, 22) FORMAT(6, 22) ' (TIME * THER DIFF) / LENGTH SQUARED = ', F5.3, 2X,
* ' TIME STEP LENGTH = ', F6.4)
N1=N-1

INITIALIZE BOUNDARY CONDITIONS

DO 30 I=1, N1
DO 30 J=1, N1
DO 30 K=1, N1
*FOR INFINITE HEAT TRANSFER COEFFICIENT*

```
IMPLICIT REAL*8(A-H,O-Z)
REAL*8 TEMPC25),AC25),B(25),C(25),D(25)
REAL*8 M1,M2
N1=N-1
```

PRINT OUT HEADING

```
WRITE(6,200)
200 FORMAT(IX, '********* RESULTS FROM METHOD OF DOUGLAS ****** *** '
&/)
```

CALCULATE COEFFICIENTS FOR THOMAS ALGORITHM (TRI-DIAGONAL MATRIX SOLVER)

```
A(I)=0.0
B(I)=1.+TSI
C(I)=-TSI
DO 60 I=2,N
A(I)=-.5*TSI
B(I)=1.+TSI
C(I)=A(I)
60
```

BEGIN TIME STEP LOOP

```
DO 20 L=1,NTOT
```

CALCULATE TEMPORARY VARIABLES "U" - X DIRECTION, "V" - Y DIRECTION, THEN ACTUAL FIELD VARIABLE "T" - Z DIRECTION.
DO 120 LOOP=1,3
DO 42 K=2,N1
DO 41 J=2,N1
DO 40 I=2,N1
IF(LOOP.EQ.2)GO TO 140
IF(LOOP.EQ.3)GO TO 141
DEGX=T(I+1,J,K)+T(I-1,J,K)-2.*T(I,J,K)
DEGY=T(I,J+1,K)+T(I,J-1,K)-2.*T(I,J,K)
DEGZ=T(I,J,K+1)+T(I,J,K-1)-2.*T(I,J,K)
M(I,J,K)=T(I,J,K)+TSI*(.5*DEGX+DEGY+DEGZ)
GO TO 40
140 CONTINUE
DELU=U(I+1,J,K)+U(I-1,J,K)-2.*U(I,J,K)
DELY=T(I,J+1,K)+T(I,J-1,K)-2.*T(I,J,K)
M(I,J,K)=M(I,J,K)+.5*TSI*(DELU-DELY)
GO TO 40
141 CONTINUE
DELV=V(I+1,J,K)+V(I,J+1,K)-2.*V(I,J,K)
DELO=T(I,J,K+1)+T(I,J,K-1)-2.*T(I,J,K)
M(I,J,K)=M(I,J,K)+.5*TSI*(DELV-DELO)
GO TO 40
40 CONTINUE
41 CONTINUE
42 CONTINUE

C INSULATED BOUNDARY ALONG X-AXIS

J=1
K=1
KP1=K+1
JP1=J+1
DO 48 I=2,N1
IP1=I+1
IM1=I-1
IF(LOOP.EQ.2)GO TO 148
IF(LOOP.EQ.3)GO TO 248
M1=.5*(T(I+1,J,K)+T(I-1,J,K)-2.*T(I,J,K))
M2=2.*(T(I,JP1,K)+2.*T(I,J,KP1)-4.*T(I,J,K))
M(I,J,K)=T(I,J,K)+TSI*(M1+M2)
GO TO 48
148 CONTINUE
DELU=U(I+1,J,K)+U(I-1,J,K)-2.*U(I,J,K)
DELY=2.*T(I,J+1,K)-T(I,J,K)
M(I,J,K)=M(I,J,K)+.5*TSI*(DELU-DELY)
GO TO 48
248 CONTINUE
DELV=2.*V(I+1,J,K)-2.*V(I,J,K)
M(I,J,K)=M(I,J,K)+.5*TSI*(DELV-2.*T(I,JP1,K)-T(I,J,K))
GO TO 48
48 CONTINUE

C INSULATED BOUNDARY ALONG Y-AXIS

I=1
IP1=I+1
K=1

KP1=K+1
DO 43 J=2,N1
JP1=J+1
JM1=J-1
IF(LOOP.EQ.2)GO TO 143
IF(LOOP.EQ.3)GO TO 243
M1=T(IP1,J,K)+T(I,JP1,K)+T(I,JM1,K)+2.*T(I,J,KP1)-5.*T(I,J,K)
M(I,J,K)=T(I,J,K)+TSI*M1
GO TO 43
143 CONTINUE
DELU=2.*(U(IP1,J,K)-U(I,J,K))
DELY=T(I,JP1,K)+T(I,JM1,K)-2.*T(I,J,K)
M(I,J,K)=M(I,J,K)+.5*TSI*(DELU-DELY)
GO TO 43
243 CONTINUE
M(I,J,K)=M(I,J,K)+.5*TSI*(DELU-DELY)
GO TO 43
43 CONTINUE

C C
INSLATED BOUNDRY ALONG Z-AXIS
C
J=1
JP1=J+1
I=1
IP1=I+1
DO 44 K=2,N1
KM1=K-1
KP1=K+1
IF(LOOP.EQ.2)GO TO 144
IF(LOOP.EQ.3)GO TO 244
M1=T(IP1,J,K)+T(I,JP1,K)+T(I,J,KM1)-5.*T(I,J,K)
M(I,J,K)=T(I,J,K)+TSI*M1
GO TO 44
144 CONTINUE
M1=M(I,J,K)
M2=M1+.5*TSI*(2.*(U(IP1,J,K)-U(I,J,K))-2.*(T(I,JP1,K)-T(I,J,K)))
M(I,J,K)=M2
GO TO 44
244 CONTINUE
DELV=T(I,J,K+1)+T(I,J,K-1)-2.*T(I,J,K)
M(I,J,K)=M(I,J,K)+.5*TSI*(DELV-2.*(T(I,J,KP1)-T(I,J,K))
44 CONTINUE

C C
INSLATED FACE AT Z=0
C
K=1
KP1=K+1
DO 45 I=2,N1
IP1=I+1
IM1=I-1
DO 45 J=2,N1
JP1=J+1
JM1=J-1
IF(LOOP.EQ.2)GO TO 145

IF(LOOP.EQ.3)GO TO 245
M1=.5*(T(IP1,J,K)+T(IM1,J,K)-2.*T(I,J,K))
M2=T(I,J,JP1,K)+T(I,JM1,K)+2.*T(I,J,KP1)-4.*T(I,J,K)
M(I,J,K)=T(I,J,K)+TSI*(M1+M2)
GO TO 45
145 CONTINUE
DELU=U(I+1,J,K)+U(I-1,J,K)-2.*U(I,J,K)
DELY=T(I,J+1,K)+T(I,J-1,K)-2.*T(I,J,K)
M(I,J,K)=M(I,J,K)+.5*TSI*(DELU-DELY)
GO TO 45
245 CONTINUE
DELV=V(I,J+1,K)+V(I,J-1,K)-2.*V(I,J,K)
M(I,J,K)=M(I,J,K)+.5*TSI*(DELV-2.*T(I,J,KP1)-T(I,J,K))
45 CONTINUE

AT THE FACE WHERE Y=0

J=1
JP1=J+1
DO 46 I=2,N1
IP1=I+1
IM1=I-1
DO 46 K=2,N1
KP1=K+1
KM1=K-1
IF(LOOP.EQ.2)GO TO 146
IF(LOOP.EQ.3)GO TO 246
M1=.5*(T(IP1,J,K)+T(IM1,J,K)-2.*T(I,J,K))
M(I,J,K)=T(I,J,K)+TSI*(M1+M2)
GO TO 46
146 CONTINUE
DELU=U(I+1,J,K)+U(I-1,J,K)-2.*U(I,J,K)
M(I,J,K)=M(I,J,K)+.5*TSI*(DELU-2.*T(I,J,KP1)-T(I,J,K))
GO TO 46
246 CONTINUE
DELZ=T(I,J,K+1)+T(I,J,K-1)-2.*T(I,J,K)
M(I,J,K)=M(I,J,K)+.5*TSI*(2.*V(I,J,JP1,K)-V(I,J,K))-DELZ)
46 CONTINUE

AT THE FACE WHERE X=0

I=1
IP1=I+1
DO 47 J=2,N1
JP1=J+1
JM1=J-1
DO 47 K=2,N1
KP1=K+1
KM1=K-1
IF(LOOP.EQ.2)GO TO 147
IF(LOOP.EQ.3)GO TO 247
M1=T(IP1,J,K)+T(I,J,JP1,K)+T(I,J,KP1)+T(I,J,KM1,K)+T(I,JM1,K)
M(I,J,K)=T(I,J,K)+TSI*(M1-5.*T(I,J,K))
GO TO 47
CONTINUE
DELY=T(I, J+1, K)+T(I, J-1, K)-2.*T(I, J, K)
M(I, J, K)=M(I, J, K)+.5*TSI*(2.*(U(IP1, J, K)-U(I, J, K))-DELY)
GO TO 47
CONTINUE
DELV=V(I, J+1, K)+V(I, J-1, K)-Z.*V(I, J, K)
DELZ=T(I, J, K+1)+T(I, J, K-1)-2.*T(I, J, K)
M(I, J, K)=M(I, J, K)+.5*TSI*(DELV-DELZ)
GO TO 47
CONTINUE
CORNER AT ORIGIN
IF(LOOP.EQ.2) GO TO 151
IF(LOOP.EQ.3) GO TO 251
M1=T(2,1,1)+2.*T(1,2,1)+2.*T(1,1,2)-5.*T(1,1,1)
M(1,1,1)=T(1,1,1)+TSI*M1
GO TO 51
CONTINUE
M1=M(1,1,1)
M2=M1+.5*TSI*(2.*(U(2,1,1)-U(1,1,1))-2.*(T(1,2,1)-T(1,1,1)))
M(1,1,1)=M2
GO TO 51
CONTINUE
M1=M(1,1,1)
M2=M1+.5*TSI*(2.*(V(1,2,1)-V(1,1,1))-2.*(T(1,1,2)-T(1,1,1)))
M(1,1,1)=M2
CONTINUE
IF(LOOP.EQ.2) GO TO 130
IF(LOOP.EQ.3) GO TO 230
C
DO 70 K=1,N1
DO 70 J=1,N1
DO 30 I=1,N1
30 D(I)=M(I, J, K)
C
CALL TRI DIAGONAL MATRIX ALGORITHM
CALL TRIDAG(1,N1,A,B,C,D,TEMP)
C
DO 430 I=1,N1
430 U(I, J, K)=TEMP(I)
C
GO TO 330
CONTINUE
DO 71 K=1,N1
DO 71 I=1,N1
DO 72 J=1,N1
72 D(J)=M(I, J, K)
C
CALL TRIDAG(1,N1,A,B,C,D,TEMP)
DO 472 J=1,N1
V(I,J,K)=TEMP(J)
CONTINUE
GO TO 330
230 CONTINUE
DO 73 I=1,N1
DO 74 J=1,N1
DO 74 K=1,N1
D(K)=M(I,J,K)
CALL TRIDAG(I,H1,A,B,C,D,TEMP)
DO 474 K=1,N1
T(I,J,K)=TEMP(K)
CONTINUE
330 CONTINUE
ITMAX=ITMAX+1
IF(ITMAX.LT.IPR)GO TO 20
WRITE(6,5)
5 FORMAT(/)
WRITE(6,91)L
91 FORMAT(1X,'TIME STEP NUMBER=',I3/)
TIME=DT*DFLOAT(L)
WRITE(6,32)TIME
32 FORMAT(5X,'ELAPSED TIME=',F10.4,'SECONDS'/)
PRINT OUT THE DIAGONAL RESULTS
WRITE(6,82)(T(I,I,I),I=1,N)
82 FORMAT(11(2X,F8.6))
ITMAX=0
20 CONTINUE
RETURN
END
SUBROUTINE TRIDAG
SUBROUTINE TRIDAG(IF,L,A,B,C,D,V)
IMPLICIT REAL*8(A-H,O-Z)
REAL*8 A(25),B(25),C(25),D(25),V(25),BETA(25),GAMMA(25)
COMPUTE INTERMEDIATE ARRAYS BETTA AND GAMMA
BETA(IF)=B(IF)
GAMMA(IF)=D(IF)/BETA(IF)
IFP1=IF+1
DO 1 I=IFP1,L
BETA(I)=B(I)-A(I)*C(I-1)/BETA(I-1)
GAMMA(I)=(D(I)-A(I)*GAMMA(I-1))/BETA(I)
COMPUTE FINAL SOLUTION VECTOR V

V(L) = GAMMA(L)
LAST = L - IF
DO 2 K = 1, LAST
I = L - K
2  V(I) = GAMMA(I) - C(I)*V(I+1)/BETA(I)
RETURN
END
WRITTEN BY R.F. HANDSCHUH

SOURCE.BURGER

***** PROGRAM #12 *****

THIS PROGRAM IS FOR THE SOLUTION OF BURGER'S EQUATION
BY THE EXPONENTIAL FINITE DIFFERENCE METHOD.

IMPLICIT REAL*8(A-H,O-Z)
REAL*8 VCI00)

READ IN DATA TO BE USED IN THE SOLUTION

WRITE(6,15)
15 FORMAT(1X,'NUMBER OF NODES=I N I3')
READ(9,10)N
10 FORMAT(I3)
WRITE(6,12)
12 FORMAT(1X,'NUMBER OF TIME SUB INTERVALS= MS I3')
READ(9,13)MS
13 FORMAT(I3)
WRITE(6,16)
16 FORMAT(1X,'TOTAL NUMBER OF TIME STEPS= MTOT I3')
READ(9,21)MTOT
21 FORMAT(I3)
WRITE(6,24)
24 FORMAT(1X,'INPUT TIME / LENGTH SQUARED  F5.3')
READ(9,25)ISI
25 FORMAT(F5.3)
WRITE(6,26)
26 FORMAT(1X,'INPUT KINEMATIC VISCOSITY= F5.3')
READ(9,27)RNU
27 FORMAT(F5.3)
WRITE(6,22)
22 FORMAT(1X,'TOTAL TIME OF ONE TIME STEP= T F5.3')
READ(9,23)T
23 FORMAT(F5.3)
WRITE(6,14)
14 FORMAT(1X,'INPUT NUMBER OF STEPS BETWEEN PRINTS/I3')
READ(9,17)IPR
17 FORMAT(I3)

DATA FOR INITIAL AND BOUNDARY CONDITIONS

V(1)=0.
V(N)=1.
N1=N-1
DO 30 I=2,N1
30 V(I)=1.

CALL EXPONENTIAL FINITE DIFFERENCE FOR BURGER'S EQUATION
CALL BURG(N,NS,NTOT,TSI,V,T,RNU,IPR)
STOP
END

SUBROUTINE BURG
SUBROUTINE BURG(N,NS,NTOT,TSI,V,T,RNU,IPR)

IMPLICIT REAL*8(A-H,O-Z)
REAL*8 VTCIOO),V(lOO),MCIOO),PCIOO),THECIOO)

TS=TSI/DFLOAT(NS+l)
DX=l.0/DFLOAT(N-1)
Nl=N-l
NSl = NS+l
WRITE(6,900)
900 FORMAT(/'***** SOLUTION FOR BURGER EQUATION *****'/)

TOTAL TIME STEP LOOP
DO 20 L=1,NTOT
ITMAX=ITMAX+1

ZERO THE SUM OF DRIVE NUMBERS
DO 15 I=1,N
PI(I)=O.
15

SET THE TEMPOARY FIELD VARIABLE EQUAL TO THE LAST TIME STEP VALUE
DO 10 I=1,N
VT(I)=V(I)
10

SUB TIME INTERVAL
DO 30 K=1,NS1

CALCULATE THE SUB-INTERVAL DRIVE NUMBERS
DO 40 I=2,N1
IM1=I-1
IP1=I+1
IF(VT(I).LE.0.0)GO TO 40
M(I)=0.5*DX*(1.-VT(I))*(VT(IP1)-VT(IM1))/VT(I)
40 CONTINUE

CALCULATE THE SUB-INTERVAL DEPENDENT VARIABLES
DO 50 II=2,N1
CHECK=TS*M(II)
IF(CHECK.LE.-50.)VT(II)=0.0
50...

VT(I1)=VT(I1)*DEXP(TS*M(I1))

CONTINUE

SUM THE DRIVE NUMBERS

DO 60 I=2,N1
P(I)=P(I)+M(I)
60 CONTINUE

CALCULATE THE DEPENDENT VARIABLE AT THE NEXT COMPLETE STEP

DO 70 I=1,N
CHECK=TS*P(I)
IF(CHECK.LE.-50.)V(I)=0.0
IF(CHECK.LE.-50.)GO TO 70
V(I)=V(I)*DEXP(TS*P(I))
70 CONTINUE

OUTPUT THE RESULTS

IF(ITMAX.LT.IPR)GO TO 20
ITMAX=0
WRITE(6,5)
5 FORMAT(/)
WRITE(6,31)L
31 FORMAT(1X,'TIME STEP NUMBER=',I3)
TIME=T*DFLOAT(L)
WRITE(6,32)L
32 FORMAT(5X,'ELAPSED TIME=',F10.4,' SECONDS')
ISTEP=(N-1)/10
DO 110 I=1,N
110 THE(I)=1.0-V(I)
DO 80 J=1,ISTEP
IS=(J-1)*11+1
IFIN=J*10+1
WRITE(6,81)(THE(I),I=IS,IFIN)
81 FORMAT(1X,11(F8.6,2X))
80 CONTINUE
20 CONTINUE
RETURN
END
WRITTEN BY R.F. HANDSCHUH

SOURCE.EXBURG

***** PROGRAM #13 *****

THIS PROGRAM IS FOR THE SOLUTION OF BURGER'S EQUATION USING AN EXPLICIT
TECHNIQUE. THE RESULTS WILL BE USED TO COMPARE TO THE EXPONENTIAL
FINITE DIFFERENCE TECHNIQUE.

IMPLICIT REAL*8(A-H,O-Z)
REAL*8 V(100)

INPUT PROGRAM DATA

WRITE(6,15) FORMAT(1X,'NUMBER OF NODES=N  I3')
READ(9,10)N
WRITE(6,16) FORMAT(I3)
READ(9,21)NTOT
WRITE(6,24) FORMAT(1X,'TOTAL NUMBER OF TIME STEPS= NTOT  I3')
WRITE(6,26) FORMAT(F5.3)
READ(9,25)TSI
WRITE(6,27) FORMAT(F5.3)
READ(9,23)RNU
WRITE(6,29) FORMAT(F5.3)
READ(9,27)T
WRITE(6,28) FORMAT(F5.3)
READ(9,26)IPR
WRITE(6,14) FORMAT(I3)

INITIALIZE THE BOUNDARY CONDITIONS

V(1)=1.
V(N)=0.
N1=N-1
DO 30 I=2,N1
V(I)=0.
30

CALL EXPLICIT FINITE DIFFERENCE SOLUTION FOR BURGER'S EQUATION

CALL BURG(N,NTOT,TSI,V,T,RNU,IPR)
STOP
END

SUBROUTINE BURG
SUBROUTINE BURG(N,NTOT,TSI,V,T,RNU,IPR)

IMPLICIT REAL*8(A-H,O-Z)
REAL*8 VTCIOO),VCIOO),THECIOO)

PRINT HEADING
WRITE(6,999)
999 FORMAT(1X,'**** EXPLICIT BURGER S EQT SOLUTION ****/)
        DX=1.0/DFLOAT(N-1)
        N1=N-1

TIME STEP LOOP
DO 20 I=1,NTOT
   ITMAX=ITMAX+1
   SAVE LAST TIME STEP VALUES
   DO 10 I=1,N
      VT(I)=V(I)
      10 CONTINUE
   EVALUATE EXPLICIT FINITE DIFFERENCE EQUATION
   DO 40 I=2,N1
      IM1=I-1
      IP1=I+1
      V(I)=VT(I)-VT(IM1)*T*(VT(IP1)-VT(IM1))/(2.*DX)
         V(I)=V(I)+RNU*T*(VT(IP1)+VT(IM1)-2.*VT(I))/(DX*DX)
   40 CONTINUE
   WRITE OUT THE RESULTS
   IF(ITMAX.LT.IPR)GO TO 20
   ITMAX=0
   WRITE(6,5)
   5 FORMAT(/)
   WRITE(6,31)L
   31 FORMAT(1X,'TIME STEP NUMBER=',I3)
   TIME=T*DFLOAT(L)
   WRITE(6,32)LTIME
   32 FORMAT(5X,'ELAPSED TIME=',F10.4,'SECONDS')
   ISTEP=(N-1)/10
   DO 110 I=1,N
      THE(I)=V(I)
      DO 80 J=1,ISTEP
         IS=(J-1)*11+1
         IFIN=J*10+1
         WRITE(6,81)(THE(I),I=IS,IFIN)
81 FORMAT(1X,11(F8.6,2X))
80 CONTINUE
20 CONTINUE
RETURN
END
SOURCE: NONBOU

WRITTEN BY R.F. HANDSCHUH

***** PROGRAM #14 *****


IMPLICIT REAL*8(A-H,O-Z)
REAL*8 (U(101),T(101),V(101))

INPUT PROGRAM DATA

WRITE(6,15)
15 FORMAT(1X,'NUMBER OF NODES IN Y DIRC=N I3'/) READ(9,10)N
10 FORMAT(I3)
WRITE(6,12)
12 FORMAT(1X,'NUMBER OF SUB INTERVALS= NS I3') READ(9,13)NS
13 FORMAT(I3)
WRITE(6,16)
16 FORMAT(1X,'TOTAL NUMBER OF X-DIR STEPS= NTOT I3') READ(9,21)NTOT
21 FORMAT(I3)
WRITE(6,24)
24 FORMAT(1X,'INPUT STEP LENGTH F5.3') READ(9,25)DX
25 FORMAT(F5.3)
WRITE(6,26)
26 FORMAT(1X,'NUMBER OF STEPS BEFORE PRINTING= I3') READ(9,27)IPR
27 FORMAT(I3)
WRITE(6,110)
110 FORMAT(1X,'INPUT KINEMATIC VISCOSITY= F6.4') READ(9,111)RNU
111 FORMAT(F6.4)
WRITE(6,101)
101 FORMAT(1X,'INPUT THERMAL DIFFUSIVITY F6.4') READ(9,102)RAL
102 FORMAT(F6.4)
WRITE(6,103)
103 FORMAT(1X,'INPUT YMAX F5.1') READ(9,104)YMAX
104 FORMAT(F5.1)
WRITE(6,250)N,NS,NTOT
250 FORMAT(1X,'# OF NODES=',I3,2X,'# OF SUB-INT=',I3,2X,

*'# OF TIME STEPS=',I3)
RN=DFLOAT(N-1)
DY=YMAX/RN
WRITE(6,251)DX,DY
251 FORMAT(1X,'DX=',F8.4,2X,'DY=',F8.4/)
WRITE(6,252)RNU,RA
252 FORMAT(1X,'KINEMATIC VISCOSITY=',F6.4,'CM*CM/S',2X,'THERMAL DIFFUSIVITY=',F6.4,'CM*CM/S')

INITIALIZE BOUNDARY CONDITIONS
DO 30 J=2,N
20 U(J)=1.0
30 V(J)=0.0
T(J)=1.0
U(1)=0.0
T(1)=0.0
CALL NON1CN(NS,NTOT,RNU,U,V,T,IPR,DX,DY,RAL)
STOP
END

SUBROUTINE NON1CN(NS,NTOT,RNU,U,V,T,IPR,DX,DY,RAL)
IMPLICIT REAL*8(A-H,O-Z)
REAL*8 UC101),MUC101),VC101),TC101),MTC101)
REAL*8 PU(101),PT(101),UTC101),TTC101),VT(101)
REAL*8 THE(101,1000,3),UT1(101)

PRINT HEADING
WRITE(6,5)
WRITE(6,222)
222 FORMAT(1X,'*********** SOURCE.NONBOU ***********'//)
WRITE(6,223)
223 FORMAT(1X,'SOLUTION FOR BOUNDARY LAYER FLOW PAST A FLAT PLATE'//)

T=DY2=DY*DY
TS2=DX*RAL/(DY2*DFLOAT(NS+1))
TS1=DX*RNU/(DY2*DFLOAT(NS+1))
DEL=DX/DFLOAT(NS+1)
NI=N-1
NS1=NS+1
NSTEP=(N-1)/10

BEGIN TOTAL INTERVAL LOOP FOR L=1 TO NTOT STEPS
DO 20 L=1,NTOT
ZERO THE SUM OF DRIVE NUMBERS FOR THE NEXT SET OF SUB-POSITION INTERVALS
DO 15 I=1,N
PU(I)=0.0
PT(I)=0.0

SAVE LAST POSITION STEP VALUES FOR TEMPORARY VARIABLE CALCULATIONS
ON SUB-POSITION INTERVAL

DO 10 I=1,N
VT(I)=V(I)
UT(I)=U(I)
TT(I)=T(I)

SUB-POSITION INTERVAL

DO 30 K=1,NS1
CALCULATE TEMPERATURE FIELD DRIVE NUMBER

DO 41 J=2,N1
JM1=J-1
JP1=J+1
MT(J)=-VT(J)*(TT(JP1)-TT(JM1))*DY/(2.*RAL*UT(J)*TT(J))
MT(J)=MT(J)+(TT(JP1)+TT(JM1)-2.*TT(J))/(UT(J)*TT(J))
CONTINUE

CALCULATE X - DIRECTION VELOCITY DRIVE NUMBER

DO 141 J=2,N1
JM1=J-1
JP1=J+1
MU(J)=-.5*VT(J)*DY*(UT(JP1)-UT(JM1))/(RNU*UT(J)*UT(J))
MU(J)=MU(J)+(UT(JP1)+UT(JM1)-2.*UT(J))/(UT(J)*UT(J))
CONTINUE

CALCULATE TEMPERATURE, X-DIRECTION VELOCITY, AND Y-DIRECTION VELOCITY
ON THE SUB-POSITION INTERVAL

DO 50 II=2,N1
TT(II)=TT(II)*DEXP(TS*MT(II))

DO 51 I=2,N1
UT1(I)=UT(I)
UT(I)=UT(I)*DEXP(TS1*MU(I))

DO 65 J=2,N1
JM1=J-1
VT(J)=VT(JM1)-.5*(DY/DEL)*(UT(J)-UT1(J)+UT(JM1)-UT1(JM1))
CONTINUE

SUM THE DRIVE NUMBERS

DO 60 J=2,N1
PU(J)=PU(J)+MU(J)
C 60 PT(J)=PT(J)+MT(J)
CONTINUE
C
C CALCULATE THE NEXT TOTAL POSITION STEP VALUES OF VELOCITIES AND TEMPERATURE
C
DO 70 J=1,N
UT(J)=U(J)
U(J)=U(J)*DEXP(TS1*F(U(J)))
T(J)=T(J)*DEXP(TS*PT(J))
CONTINUE
C
DO 75 J=2,N1
JM1=J-1
V(J)=V(JM1)-.5*(DY/DX)*(U(J)-UT(J)+U(JM1)-UT(JM1))
ITMAX=ITMAX+1
C
SAVE THE VALUES FOUND IN 3-DIMENSIONAL ARRAY "THE"
C
DO 76 J=1,N
THE(J,L,1)=U(J)
THE(J,L,2)=V(J)
THE(J,L,3)=T(J)
IF(ITMAX.LT.IPR)GO TO 20
C
WRITE OUT THE RESULTS AT THE REQUESTED INTERVAL OF POSITION
C
WRITE(6,5)
FORMAT(/)
WRITE(6,31)L
5 FORMAT(5X,'POSITION STEP NUMBER=',I3)
TSTEP=DX*DFLOAT(L)
WRITE(6,32)TSTEP
31 FORMAT(5X,'X-POSITION=',F10.4)
WRITE(6,101)
101 FORMAT(1X,'THE U VELOCITY COMPONENT')
DO 300 KK=1,NSTEP
IS=(KK-1)*11+1
IFIN=KK*10+KK
WRITE(6,82)(THE(I,L,1),I=IS,IFIN)
300 CONTINUE
82 FORMAT(11(2X,F8.5))
WRITE(6,102)
102 FORMAT(1X,'THE V VELOCITY COMPONENT')
DO 301 KK=1,NSTEP
IS=(KK-1)*11+1
IFIN=KK*10+KK
WRITE(6,82)(THE(I,L,2),I=IS,IFIN)
301 CONTINUE
103 FORMAT(1X,'THE T FIELD VARIABLE ')
DO 302 KK=1,NSTEP
IS=(KK-1)*11+1
IFIN=KK*10+KK
WRITE(6,103)
103 FORMAT(1X,'THE T FIELD VARIABLE ')
WRITE(6,82)(THE(I,L,3),I=IS,IFIN)

CONTINUE
ITMAX=0
CONTINUE
RETURN
END
An exponential finite difference technique for solving partial differential equations has been extended. The finite difference algorithm developed was used to solve the diffusion equation in one-dimensional cylindrical coordinates and applied to two- and three-dimensional problems in Cartesian coordinates. The method was also used to solve nonlinear partial differential equations in one (Burger's equation) and two (Boundary Layer equations) dimensional Cartesian coordinates. Predicted results were compared to exact solutions where available, or to results obtained by other numerical methods. It was found that the exponential finite difference method produced results that were more accurate than those obtained by other numerical methods, especially during the initial transient portion of the solution. Other applications made using the exponential finite difference technique included unsteady one-dimensional heat transfer with temperature varying thermal conductivity and the development of the temperature field in a laminar Couette flow.