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# Orbital Transfer Rocket Engine Technology Program Integrated Control And Health Management

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Phase II Task E.3 Final Report  
Contract NAS 3-23772  
NASA CR 182122  
October 1988

Prepared For:  
National Aeronautics and Space Administration  
Lewis Research Center  
Cleveland, Ohio 44135

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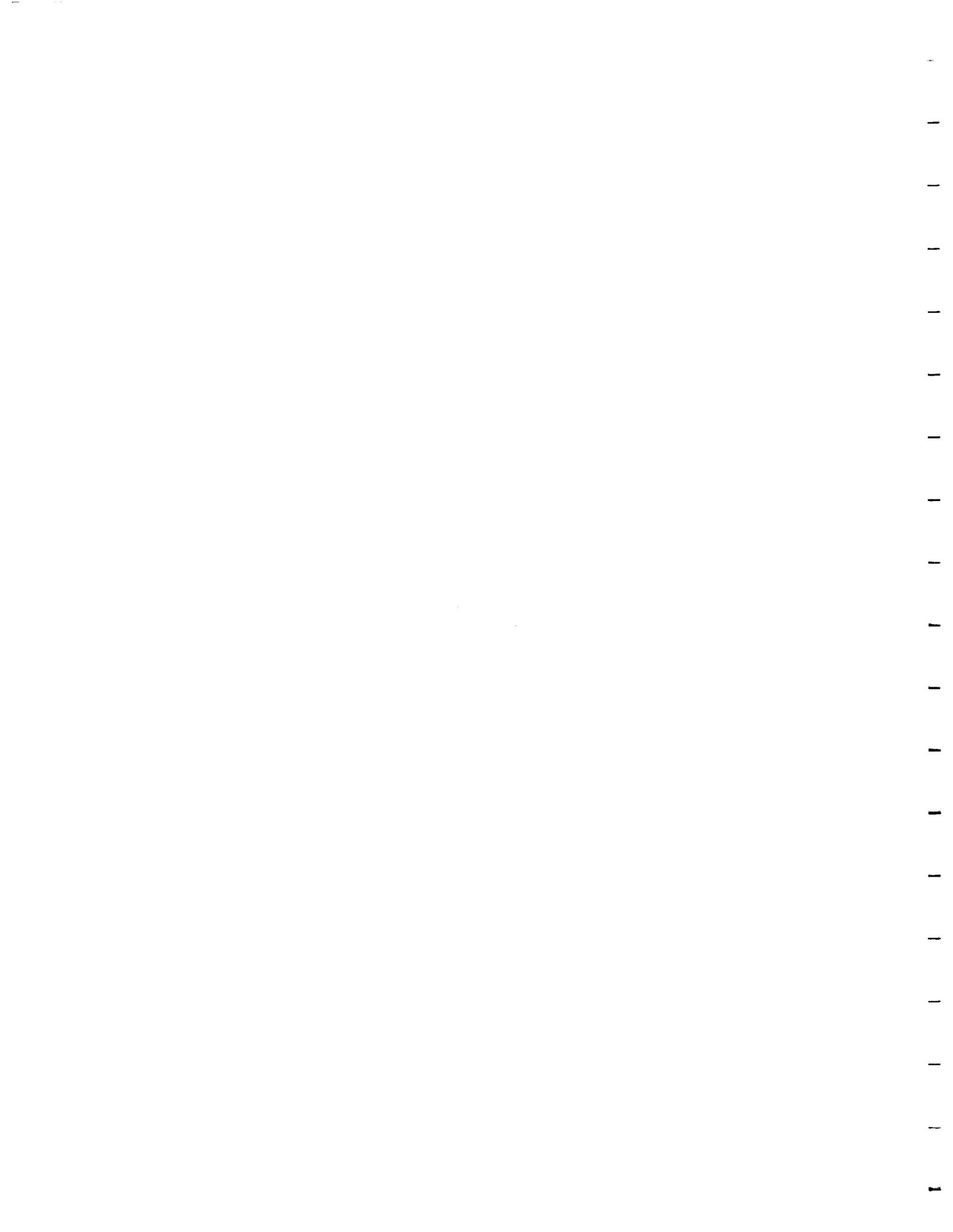
INTEGRATED CONTROL AND HEALTH MANAGEMENT  
PHASE II TASK FINAL REPORT  
ORBIT TRANSFER ROCKET ENGINE TECHNOLOGY PROGRAM

Prepared For  
National Aeronautics and Space Administration

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NASA Lewis Research Center  
Contract NAS 3-23772

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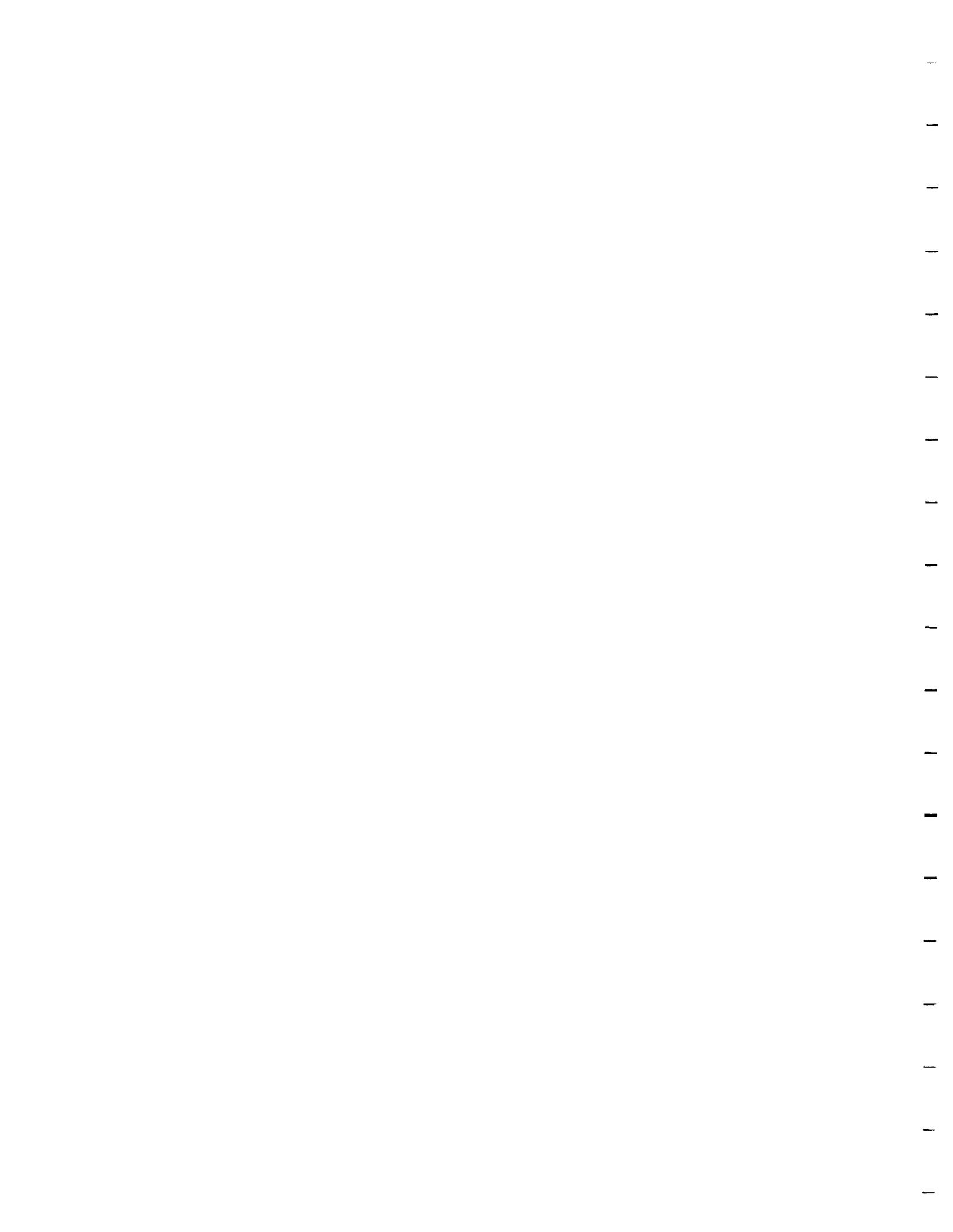


TABLE OF CONTENTS

|   | <u>Page</u> |
|---|-------------|
| I. Summary  | 1           |
| II. Introduction  | 3           |
| A. Background   | 3           |
| B. Scope  | 12          |
| III. Engine System Modeling                                       | 14          |
| A. Control Requirements   | 14          |
| B. Control Valves Used in the Engine Model                        | 14          |
| C. Model Building   | 14          |
| D. Start Modes/Throttling   | 19          |
| IV. Control Logic Evaluation                                      | 29          |
| A. System Gains and Shaping Filters                               | 29          |
| B. Control System Stability Analysis                              | 29          |
| C. System Performance   | 35          |
| D. Control Model Updates  | 35          |
| E. Control Configuration  | 36          |
| V. Control Element Requirements                                   | 41          |
| A. Control Valve Requirements                                     | 41          |
| B. Control Sensor Requirements                                    | 41          |
| C. Engine Controller Requirements                                 | 41          |
| VI. Health Management   | 49          |
| A. Health Management System Description                           | 49          |
| B. Engine Failure Modes   | 49          |
| C. Designing for System Reliability                               | 53          |
| D. Modular Health Management System Design                        | 57          |
| E. Health Management Data Storage                                 | 57          |
| F. Applications for Artificial Intelligence and<br>Expert Systems | 64          |
| G. Health Management Systems                                      | 65          |
| H. Sensor Supplier Surveys  | 71          |
| I. Review of Vehicle Prime Studies                                | 77          |
| VII. Discussion of Results  | 85          |
| A. Engine Control and Modeling                                    | 85          |
| B. Control Elements   | 86          |

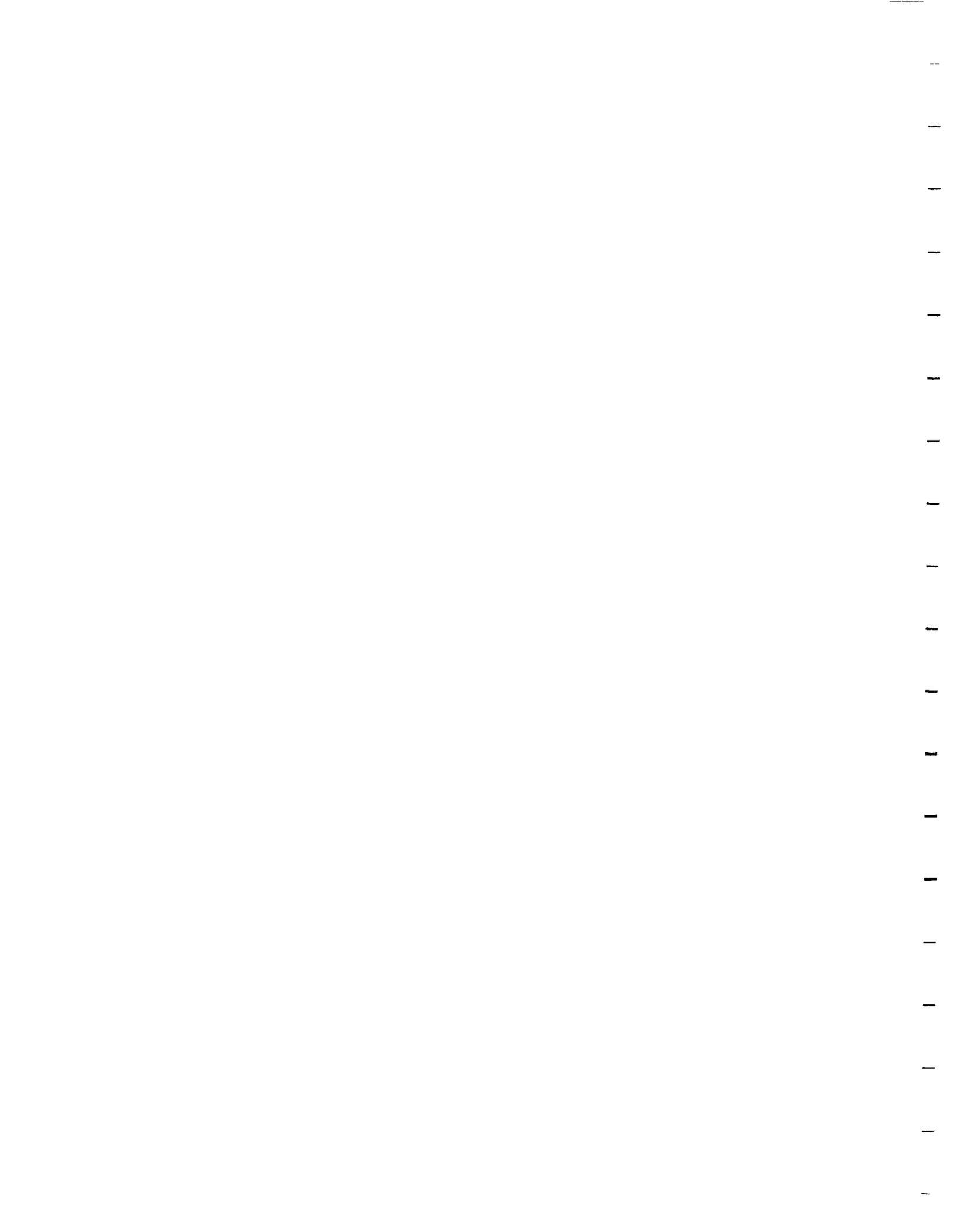
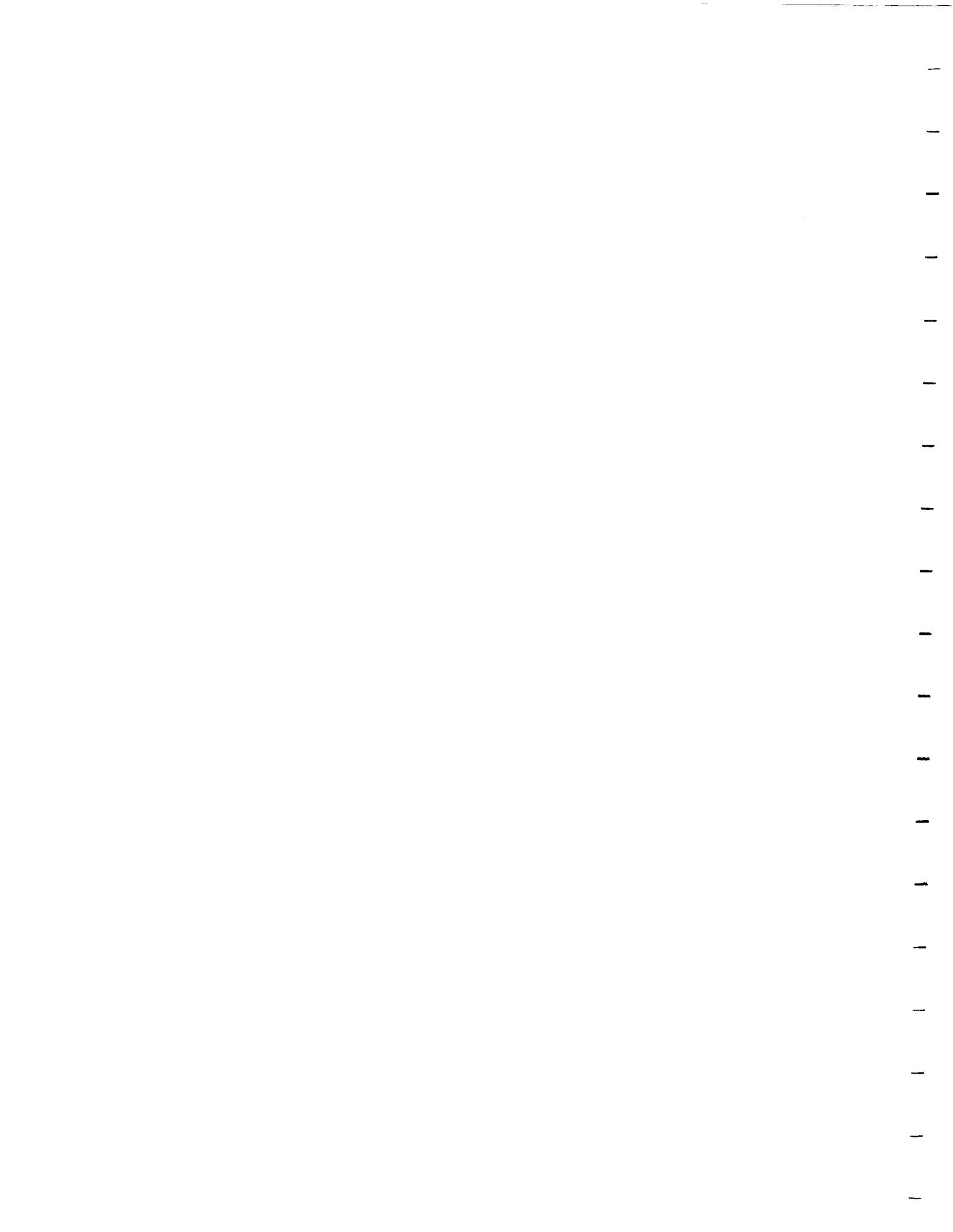


TABLE OF CONTENTS (cont.)

|   | <u>Page</u> |
|---|-------------|
| C. Health Monitoring                                    | 87          |
| VIII. Summary of Results                                | 88          |
| IX. References  | 91          |
| Appendices  |             |
| A. Sensor Supplier Survey Results                       | A-1         |
| B. Preliminary Valve and Sensor Specifications          | B-1         |
| C. OTV Engine Failure Modes and Effects Analysis (FMEA) | C-1         |



LIST OF TABLES

| <u>Table No.</u> |  | <u>Page</u> |
|------------------|--|-------------|
| I                | OTV Engine Valves and Sensors for Engine Control                             | 11          |
| II               | OTV Engine Design Drivers  | 15          |
| III              | Symbols Used in the Simulation and Control Models                            | 20          |
| IV               | Parameters in the Engine Linear Dynamic Model                                | 33          |
| V                | OTV Engine Control Valves  | 42          |
| VI               | Engine Sensors for Baseline Control Logic                                    | 44          |
| VII              | Selected Control Logic, Sensors and Valves Drive the Controller Requirements | 45          |
| VIII             | IR&D Engine Controller - Engine Interfaces                                   | 46          |
| IX               | OTV Engine Failure Modes   | 51          |
| X                | Health Management Algorithm  | 58          |
| XI               | Health Management Software at Engine Level                                   | 59          |
| XII              | Health Management Software at Sensor Level                                   | 60          |
| XIII             | Available Data Storage Technologies  | 63          |
| XIV              | Control and Health Management Sensors  | 66          |
| XV               | Displacement Sensor Requirements   | 68          |
| XVI              | OTV Engine Space Maintainable Components                                     | 79          |
| XVII             | Initial OTV Delivery and Assembly Operational Functions                      | 80          |
| XVIII            | OTV Servicing Operational Functions  | 81          |
| XIX              | Space-Based OTV Maintenance Functions  | 82          |
| XX               | OTV Engine Removal Operations  | 83          |



## LIST OF FIGURES

| <u>Figure No.</u> |   | <u>Page</u> |
|-------------------|---|-------------|
| 1                 | Aerojet's Approach to ICHM  | 4           |
| 2                 | Basic Design Strategy for ICHM Sensors  | 5           |
| 3                 | Sensor Development Challenges   | 7           |
| 4a                | OTV Engine Control Schematic  | 9           |
| 4b                | OTV Engine Dual Expander Cycle  | 10          |
| 5                 | Engine Dynamic Simulation   | 16          |
| 6                 | TUTSIM Program Application  | 18          |
| 7                 | OTV Engine Control Configuration  | 22          |
| 8                 | Turbine Bypass Valve Fuel Schedule  | 23          |
| 9                 | OTV Engine Turbine Bypass Percent vs Chamber Pressure   | 28          |
| 10                | Predicted Response to 10% Throttle Down Command   | 30          |
| 11                | Predicted Response to 10% Throttle Up Command   | 31          |
| 12                | OTV Engine Linear Dynamic Model   | 32          |
| 13                | Selected Control Logic Provides Stable Engine Operation   | 37          |
| 14                | OTV Engine Start-up Sequence  | 38          |
| 15                | OTV Engine Shut-Down  | 39          |
| 16                | Health Management System Configuration  | 50          |
| 17                | Layered HM/C System Architecture  | 52          |
| 18                | Generalized Detection Algorithm Flow Diagram  | 54          |
| 19                | Testbed System with Advanced Diagnosis Algorithm (ADIA)   | 56          |
| 20                | Normal Mode Accommodation Filter Logic  | 56          |
| 21                | Example of a System Fault Detection/Verification Logic for a Gas Generator Turbine Drive System | 61          |
| 22                | Condition Monitoring Block Diagram  | 62          |
| 23                | Example of a Bearing Health Monitoring Silicon Sensor   | 69          |
| 24                | Reproduction of Actual Oscilloscope Traces of Turbopump Bearing Signatures                      | 70          |
| 25                | Multifunction Turbopump Shaft Monitor - Longitudinal Section                                    | 72          |
| 26                | Multifunction Turbopump Shaft Monitor - Active Tip Configuration                                | 73          |
| 27                | Hot Spot Sensor Assembly  | 74          |



LIST OF FIGURES (cont.)

| <u>Figure No.</u> |   | <u>Page</u> |
|-------------------|---|-------------|
| 28                | Hot Spot Sensor Configuration                         | 75          |
| 29                | OTV Engine Health Management - Sensor Supplier Survey | 76          |



## I. SUMMARY

When this engine control and health management task was approved engine designs for the Orbit Transfer Vehicle (OTV) had progressed to a point where a control assessment was necessary for confidence that the proposed expander cycle was practical. The oxygen and hydrogen turbopumps are mechanically separate, flow circuits are complicated, and numerous valves are required for these complex engines. Resources limited the scope of the work to developing a nonlinear dynamic model, but simulations with the model predict full range throttling, thrust, and mixture ratio control with the baseline engine controls. This success is an important validation of the engine design approach and encourages continued engine development. Additional work is needed, however, prior to full-scale engineering development to verify the linear dynamic model and to map the complete engine control range.

The second major task was to integrate the controls approach with a health management system. An integrated systems approach had been recommended by ATC and approved by the NASA as a means of reducing sensor numbers and data streams. The health management system also directly affects engine life and mission success, both important design considerations for the engine. It will also be used for pre- and post-flight health assessment. During the mission it will command the controller to prevent equipment damage or a catastrophic failure; very important functions for a piloted vehicle. The study produced a basic health management architecture fully integrated with the engine control function. Specific sensors were identified and simplified specifications developed for both sensors and the most important mechanical control elements, the valves. Enough of the groundwork has been completed to define the software development task that would make the system operational.

A major unknown in the integrated control and health management (ICHM) task was the amount of development work needed for the engine controller. This had been considered a \$5 to 10M project based on the system complexity if a controller had to be specifically designed for this engine. Aerojet was able to verify that a new controller developed for pump-fed storable propellant engines could be modified to meet the OTV engine requirements. With a controller identified all

I, Summary (cont.)

of the basic control and sensor hardware is either available or in the active development cycle. This assures that the ICHM requirements of the engine can be met within any reasonable development schedule.

The next requirement for additional ICHM development will come in the preparation for a test bed or integrated component test program. Actual sensors and control valves will be used with software developed for a specific test site but representative of that needed for the engine controller. This report is the logical starting point for that phase of the engine development.

## II. INTRODUCTION

### A. BACKGROUND

#### 1. Engine Control and Health Monitor Sensor Capability

Aerojet has investigated engine control requirements for the Orbit Transfer Vehicle Engine under NASA contracts going back to 1979 (NAS8-33574, see ref. 1). The present contract, however, departs from the former program direction by changing the emphasis from a strictly engine control investigation to an integration of engine control with engine health management. The health monitoring technology focus of present NASA interest is in sensor development. Aerojet TechSystems Co. (ATC) shares this interest and believes that a comprehensive health management system is limited primarily by the availability of suitable sensors. Fortunately, much new sensor technology is being developed for automotive, industrial and aerospace applications. ATC believes that much of this new technology is applicable to rocket engines and can and should be adapted to programs such as OTV. The ATC approach to sensor development is to search out the best available technology for the specific application, modify and adapt it as required, and team with the most qualified vendor for its development.

#### 2. Aerojet Approach to the ICHM Task

A summary chart of ATC's approach to the ICHM task is given in Figure 1. The NASA has accepted this approach to the extent that this preliminary ICHM task is largely concerned with the OTV engine control's development and modeling. With acceptable control analysis results obtained, the baselined control sensors can be used as a logical starting set for addressing health management sensor requirements. The necessary control and health management sensors have been identified, are listed in this report, and preliminary specifications have been prepared. In addition, ATC has investigated other aspects of sensor data use in the Orbit Transfer Vehicle as a means of deriving a basic design strategy for ICHM sensors that focuses on sensors having the greatest potential utility, not just those involving innovative sensing methods. Our basic design strategy is outlined in Figure 2. It is a conservative approach that is more concerned with extending the state-of-the-art rather than redefining it. The major benefits to be gained are high reliability and predictable sensor performance. A conservative approach to sensor

- ENGINE CONTROL MUST BE GIVEN FIRST PRIORITY DUE TO THE MISSION REQUIREMENTS AND UNIQUE DESIGN OF THE OTV ENGINE
  - ENGINE CONTROL IS A CRITICAL TECHNOLOGY THAT MUST BE VALIDATED FOR ANY NEW ENGINE
  - SYNCHRONIZED OPERATION OF MECHANICALLY SEPARATE TURBOPUMPS HAS BEEN DEMONSTRATED ONLY ON THE SSME
  - CONTROL REQUIREMENTS FOR THROTTLING OPERATION AND MANRATING ARE PARTICULARLY DEMANDING
  - DEFINITION OF CONTROL SENSORS IS A LOGICAL STARTING POINT FOR DEVELOPING SENSOR REQUIREMENTS FOR HEALTH MONITORING
- TRANSITION FROM ENGINE CONTROL VALIDATION TO SENSOR DEVELOPMENT
  - ATC CONSIDERS CONTROL STUDY RESULTS ACCEPTABLE BUT MORE WORK IS NEEDED
  - HEALTH MANAGEMENT SENSOR DEVELOPMENT IS CURRENT PROGRAM INTEREST
  - CONTINUATION OF ENGINE CONTROL WORK SHOULD FOLLOW ENGINE PRELIMINARY DESIGN
- SENSOR DATA MUST SATISFY USER REQUIREMENTS
  1. ENGINE CONTROL - HIGHEST PRIORITY, GREATEST REDUNDANCY
  2. ENGINE HEALTH MONITORING - BOTH MISSION AND MAINTENANCE USE
  3. PILOT STATION DISPLAY - REAL TIME PROCESSED INFORMATION
  4. VEHICLE DATA TELEMETRY - REAL TIME DATA STREAM SELECTION
- DESIGN FOR COST EFFECTIVENESS
  - MULTIPLE USE OF SENSOR DATA WHEN POSSIBLE
  - CRITICAL SENSOR TASKS REQUIRE REDUNDANCY OR INDIRECT DATA DERIVATION FROM OTHER SENSORS
  - FMEA IDENTIFIES NEEDED SENSORS AND THE SENSOR RELIABILITY REQUIREMENTS
  - SENSOR COST CAN BE A SIGNIFICANT PART OF TOTAL ENGINE COSTS
  - NEW SENSORS CAN HAVE A DEVELOPMENT TIME MEASURED IN YEARS
  - SENSOR INTERFACE/POWER/DISPLAY REQUIREMENTS EFFECT ENGINE AND VEHICLE DESIGN

Figure 1. Aerojet's Approach to ICHM

- Use proven sensor designs when possible
- Miniaturize both electronics and transducers
- Consider that ruggedness and reliability tend to go together
- Require health management sensors for rotating parts
  - Vibration/out-of-balance indicators
  - Wear indicators
  - Seal failure indication
  - Rubbing/contact indicator for hydrostatic bearings
- Emphasize development of health management sensors used for maintenance decisions and system troubleshooting
  - Valve actuation rate sensing and recording
  - Leak and/or crack detection sensors
  - Temperature anomaly detection
  - Out-of-operating range excursions: temperature, mixture ratio, thrust, pressure sensing, recording and evaluating
- Sensors for operation record
  - Pc trace, flowmeter traces, turbine inlet temp. traces, engine compartment temp. trace, etc.
  - All data digitized and time correlated on recording

Figure 2. Basic Design Strategy for ICHM Sensors

## II, A, Background (cont.)

development does not rule out innovation or free the development from challenges. Figure 3 is a summary of the challenges perceived in sensor development for the OTV engine. ATC expects to propose an ICHM task to the NASA that addresses the serious computational and processing needs arising from the immense quantity of sensor data that will be available. An application from the rapidly developing field of expert systems and other artificial intelligence (AI) techniques is one likely possibility.

### 3. Aerojet Dual Expander Engine Cycle

The ATC version of the OTV engine baselines a unique form of the expander cycle developed for high performance, turbopump-fed liquid propellant rocket engines. In a conventional hydrogen expander cycle the fuel is routed through the jacket of a regeneratively cooled engine where it acquires sufficient thermal energy to power the turbine drives of both fuel and oxidizer. The propellant then is routed to the injector and combustion chamber. This cycle is fairly simple and offers good performance potential. As all the propellant is burned in the combustion chamber there are no losses comparable to those from open cycles. Its limitations are related to dependence on only one working fluid. High chamber pressures (>1500 psia) require high (>1000°F) hydrogen exit temperatures from the regen chamber. This poses limits to materials and cycle life concerns. The hydrogen working fluid must also drive pumps for both fuel and oxidizer. This requires an interpropellant seal in the oxygen pump or a gear drive. There are also mixture ratio limitations and demanding design requirements for needed heat transfer within an acceptable chamber length.

The ATC dual expander cycle makes use of both fuel and oxidizer as working fluids. For the OTV engine the hydrogen fuel is used as the regen coolant for the combustion chamber jacket where it acquires sufficient energy to run the turbine driving the hydrogen turbopump before entering the injector. The engine concept used in the control study has an oxygen cooled nozzle extension with additional oxygen heating from a LO<sub>2</sub>/GH<sub>2</sub> heat exchanger. The combined heat input raises the oxygen to 400°F. This is sufficient to drive the turbine on the oxygen turbopump but is considered a safe temperature for the turbine materials.

- CHARACTERIZE MOVING PARTS
  - RATES, NOT POSITION CHANGE ALONE
  - EXCURSION FROM SMOOTH, VIBRATION-FREE OPERATION
  - BEARING WEAR RATE AND LOCATION
  - UNUSUAL THERMAL GRADIENTS
  
- MINIATURIZATION WITHOUT FRAGILITY
  - SENSORS BUILT INTO THE PART
  - FORMED IN TO DAMAGE RESISTANT SHAPES (SHORT, THICK VS. LONG, THIN)
  - WIRELESS DESIGNS WITHIN A COMPONENT
  - USE OF EQUIPMENT GENERATED SIGNALS (PASSIVE TRANSDUCER ELEMENTS)
  
- REDUCE COST, WEIGHT, POWER REQUIREMENTS
  - TRADE REDUNDANCY AND RELIABILITY
  - USE AI TO DEVELOP INFORMATION FROM OPERATING SYSTEMS WHEN THERE IS A FAILURE OF SOME SENSORS
  - SIGNAL PROCESSING BY MICROELECTRONICS AT THE SENSOR SITE

Figure 3. Sensor Development Challenges

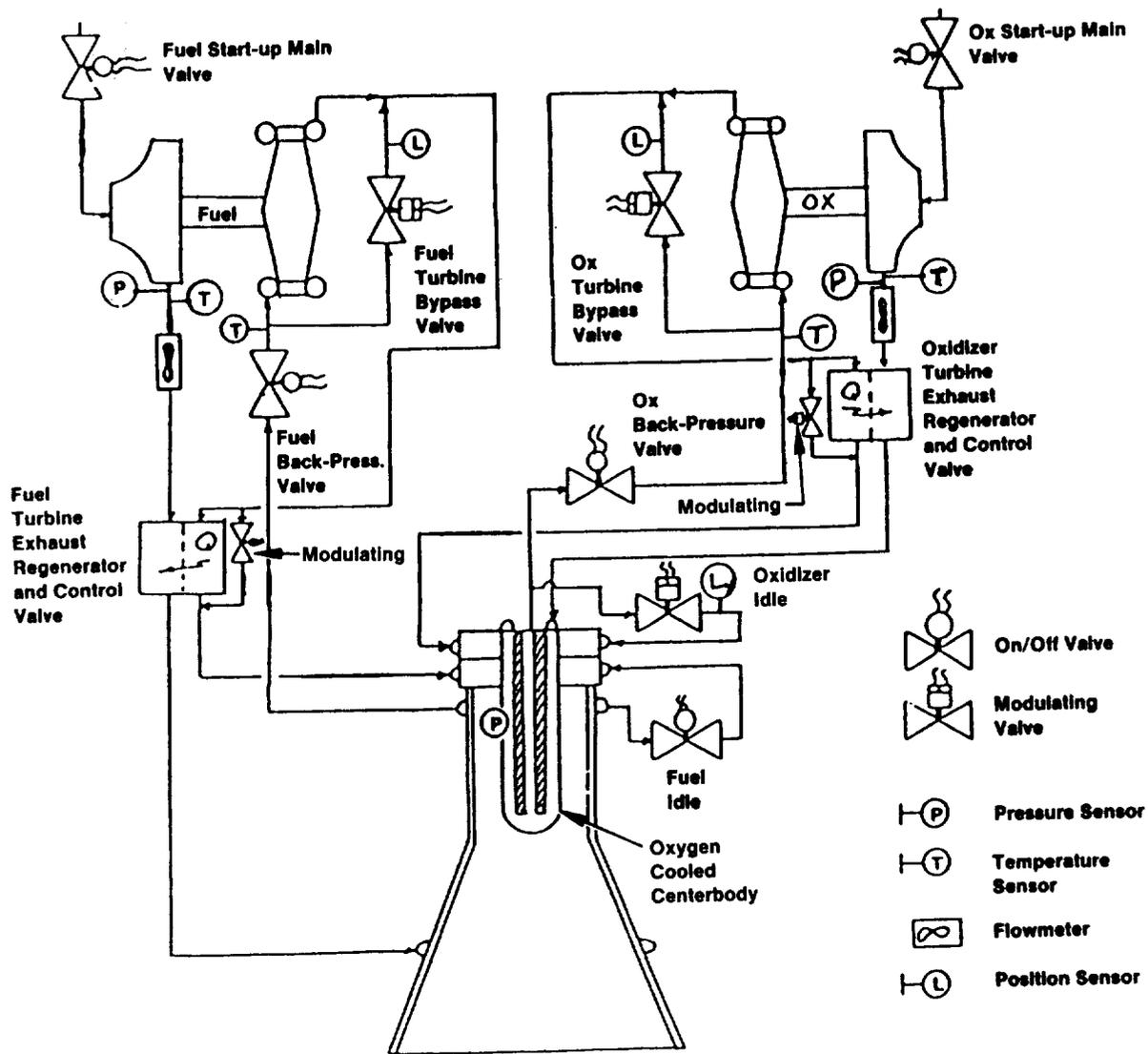
## II, A, Background (cont.)

By using both fuel and oxidizer as working fluids the dual expander cycle can deliver higher combustion chamber pressures at lower fluid temperatures than the conventional expander cycle. For throttling operation the mixture ratio limits are considerably broadened while maintaining close to optimum specific impulse. There is also a design and safety benefit from removing the need for an interpropellant seal from the oxygen turbopump. Of major importance at the system level is the capability to operate the engine without a helium purge system. This gives a major weight advantage to the cycle.

### 4. Dual Expander Cycle Engine Control

The OTV engine control schematic used for modeling is presented in Figure 4a. This control schematic was subsequently modified during the engine preliminary design task to the configuration shown in Figure 4b. Conceptually there is little difference between the two schematics but the changes in design details are substantial. The major change was to delete the oxygen cooled centerbody and provide oxygen heating by a LO<sub>2</sub>/GH<sub>2</sub> heat exchanger and the regen cooled nozzle extension. Also added were boost pumps and provisions for autogenous tank pressurization. The major control concern is the mechanically separate turbopumps. Synchronization is absolutely dependent on sensor data properly interpreted and acted upon by the engine controller and control software. All engine operations are ultimately controlled by the 11 valves shown on the schematic in Figure 4b plus the two igniter valves. Valve control functions are listed in Table I along with the sensor data used to position the valves. The idle valve and back pressure valves are used only during startup and idle mode operation. The regenerator bypass valve and LO<sub>2</sub>/GH<sub>2</sub> heat exchanger valve are used for working fluid temperature control. The turbine bypass valves control thrust and mixture ratio. All valve sequencing and positioning is done through an electronic controller or through line pressure actuation. The controller, in turn, receives all the relevant sensor input as well as operation commands. This engine has a degree of complexity beyond the limits for a mechanical start and operating procedure; a controller is necessary to its operation.

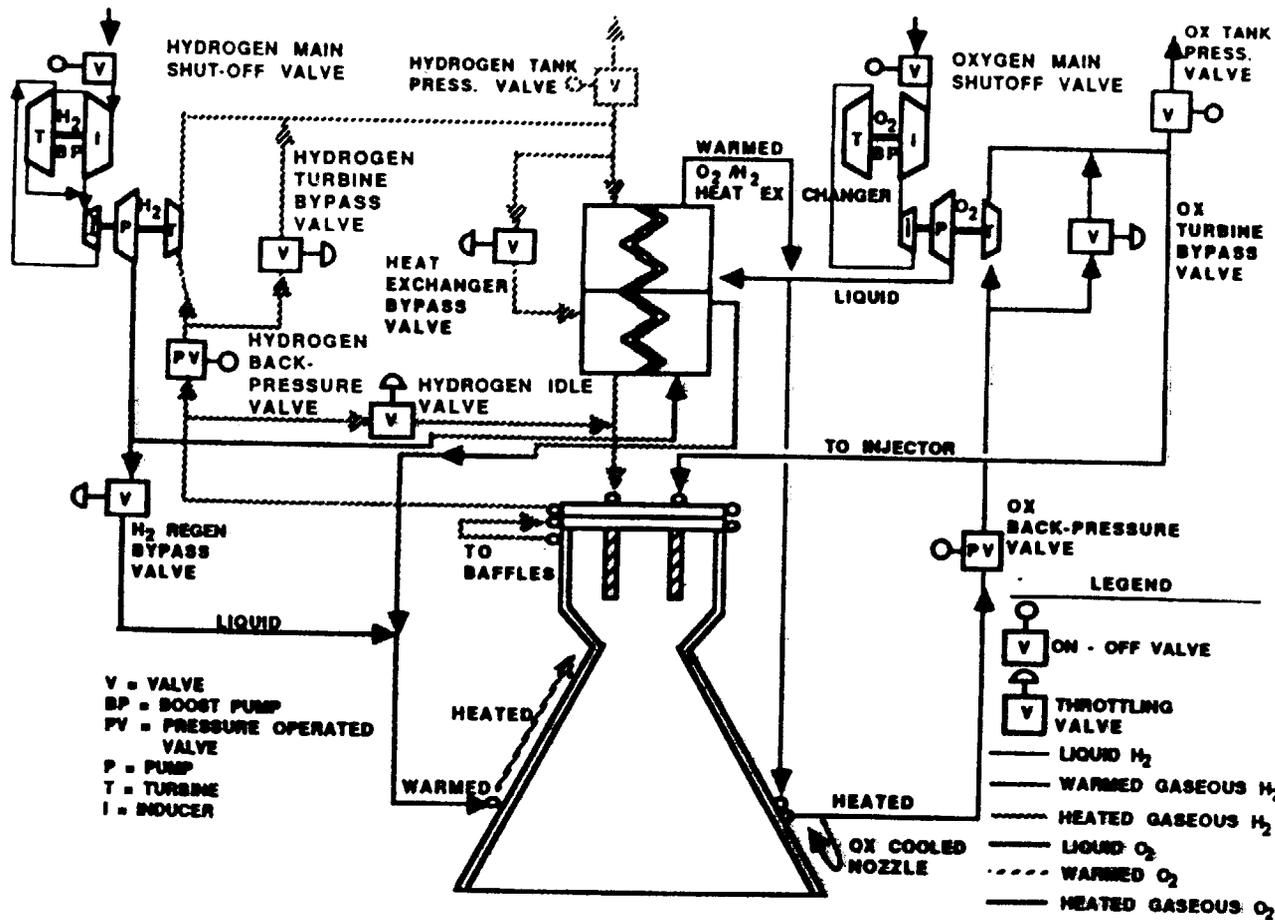
A direct spark ignition system was successfully tested in the ATC 3.0K lbf thrust TCA hot-fire test program. It is the baseline ignition system for the engine.



Nominal Thrust = 3000 lbf  
 Max. Thrust = 3750 lbs

Figure 4a. OTV Engine Control Schematic

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As of 1 May 1968  
3.1.0.48

Nominal Thrust = 7500 lbf  
Max. Thrust = 8625 lbs

Figure 4b. OTV Engine Dual Expander Cycle

TABLE I

OTV-ENGINE VALVES AND SENSORS FOR ENGINE CONTROL

| <u>Valve Name</u>            | <u>Abbreviation</u> | <u>Control Function</u>   | <u>Primary Sensor Data Source</u>                                 |
|------------------------------|---------------------|---------------------------|---|
| Hydrogen Turbine Bypass      | HTBV                | Mixture Ratio             | Hydrogen, Oxygen Flowmeters, Chamber Pressure                     |
| Oxygen Turbine Bypass        | OTBV                | Thrust Flowmeters         | Oxygen Flowmeters, Hydrogen                                       |
| Heat Exchange Bypass         | HEB                 | Oxygen Max. Temperature   | Oxygen Turbine Inlet Temperature                                  |
| Hydrogen Regenerator Bypass  | HRBV                | Hydrogen Max. Temperature | Hydrogen Turbine Inlet Temperature                                |
| Hydrogen Idle                | HIV                 | Idle Mode Start           | Flowmeter Data, Chamber Pressure                                  |
| Hydrogen Back Pressure       | HBPV                | Start Pressure Balance    | Line Pressure Operated  |
| Oxygen Back Pressure         | OBPV                | Start Pressure Balance    | Line Pressure Operated  |
| Hydrogen Main Shutoff        | HMSV                | Propellant Isolation      | Commanded by OTV Pilot (Engine Start, Stop) or Engine Controller  |
| Oxygen Main Shutoff          | OMSV                | Propellant Isolation      | Commanded by OTV Pilot (Engine Start, Stop), or Engine Controller |
| Hydrogen Igniter Control     | HICV                | Engine Start              | Chamber Pressure  |
| Oxygen Igniter Control       | OICV                | Engine Start              | Chamber Pressure  |
| Oxygen Tank Pressurization   | OTPV                | Tank Pressurization       | Tank Pressure, Chamber Pressure                                   |
| Hydrogen Tank Pressurization | HTPV                | Tank Pressurization       | Tank Pressure, Chamber Pressure                                   |

## II, Introduction (cont.)

### B. SCOPE

#### 1. Task Order Scope Definition

Provide the necessary personnel and support to continue the technology development of a health management system for this OTV engine concept. Build on earlier work to improve the engine system model, evaluate the control logic, specify and determine the availability of control components and sensors, and integrate health management into the control system.

#### 2. Specific Subtasks

##### Subtask I - Engine System Modeling

Continue engine system dynamic modeling using the baseline engine system developed under previous contract efforts. An approach is to compile an initial model from simplified models of the primary engine components (i.e., TPAs, thermal and combustion processes and control elements). Engine model sophistication is increased by including more detail for the individual component models.

Update engine models, both dynamic and steady-state, as hardware test data becomes available or as control system changes are identified in Subtask II.

##### Subtask II - Control Logic Evaluation

Evaluate the preliminary control logic identified in earlier contract work using the engine system dynamic models developed in Subtask I. Perform control system stability analyses and determine system gains and shaping filter requirements for a stable, responsive system. Perform a controllability analysis to determine the engine and control system response during transient conditions such as start-up, shut-down and throttling.

II, B, Scope (cont.)

Modify the control logic as necessary to achieve the desired system performance. Change the baseline control system if satisfactory performance cannot be achieved by adjusting the control logic algorithms.

Subtask III - Control Element Requirements

Establish control element requirements. Determine control valve parameters such as flow areas, resolution, accuracy, actuation method, valve type and operating environment. Identify control sensors required to implement the control system logic identified in Subtask II. Preliminary requirements for these sensors will be established. Determine basic engine controller requirements such as sensor processing, valve output drive capability, and computation and memory needs. For each of the control elements required to operate the engine, assess current and/or future component availability.

Subtask IV - Health Monitoring

Evaluate the baseline control system design so that an integrated control and health management design approach can be achieved. Key areas to be considered include sensors, computational requirements and algorithms. Emphasis is to be placed on the degree of health management that can be achieved using the control system sensors identified in Subtask III. Document the set of sensors, the parameters sensed or synthesized from this set, and the algorithms for synthesizing these parameters. The benefits of additional sensors to enhance health monitoring capabilities shall be considered. Review of prime contractor OTV studies so that the requirements for space-based repair/maintenance, modularity or other appropriate factors, such as life cycle cost, are accounted for in the health monitor design.

### III. ENGINE SYSTEM MODELING

#### A. CONTROL REQUIREMENTS

A throttling range of 10:1 and mixture ratio variation of 5.0 to 7.0 over this range has been specified for the OTV engine. The nominal operating design point assumes a thrust of 7500 lbf and mixture ratio of 6.0. A dual expander engine cycle has the capability of meeting the throttling requirements within the limitations of the mixture ratio range. This is important in maintaining near optimal specific impulse from minimum to maximum thrust. A list of engine requirements is given in Table II.

#### B. CONTROL VALVES USED IN THE ENGINE MODEL

Several control valve configurations were investigated in previous studies in order to determine their effect on engine control. This was achieved by using power balance models to determine the sensitivity of engine thrust and mixture ratio to valve perturbations. The in-line throttling concepts were determined to be unsuitable as they exhibited practically no control over mixture ratio. Analytical results showed that bypass valves at the oxidizer and fuel turbines are effective in meeting the control objectives. Thrust control is provided by simultaneous modulation of the oxygen and hydrogen turbine bypass valves. Mixture ratio adjustments are made using the hydrogen turbine bypass valve. One of the important functions of the controller is to maintain stable engine operation. This would be impossible to achieve in an open-loop system due to the coupling between thrust and mixture ratio.

#### C. MODEL BUILDING

##### 1. Dynamics Simulation Model

The dynamics model shown in Figure 5 was used to study the transient behavior of the engine system. TUTSIM<sup>®</sup>, a dynamics simulation package, was implemented for the actual model coding and simulation work. TUTSIM<sup>®</sup> is an acronym based on Twente University of Technology Simulation Software. It is a product of the software firm Applied I of Palo Alto, California. It was designed for use on the IBM PC/XT/AT/Jr computers. A number of configurations are available

**Table II  
OTV Engine Design Drivers**

| Driver                        | Measure of Merit   | Goal/Requirement  |
|-------------------------------|--|---|
| 1. Performance                | Specific Impulse   | 490 Seconds   |
| 2. Weight                     | Dry Weight of Deliverable Eng  | 180 lbm   |
| 3. Cycle Life                 | #Starts, Hours of Operation,<br>Hot Gas Side Wall Temp.,<br>Operating Envelope               | 100 Starts, 4 Hours Service<br>500 Starts, 20 Hours Opera-<br>tional Life |
| 4. Gimbal                     | Pitch & Yaw Movement,<br>Weight, Envelope  | ± 20 Degrees  |
| 5. Packaging                  | Engine Length with Nozzle<br>Stowed, Vehicle Interface                                       | TBD, Assume 60 Inches   |
| 6. Combustion Stability       | Steady State Thrust Variation,<br>Percent Over Operating Range                               | Not Specified   |
| 7. Thrust                     | #Elements, Thrust/Element,<br>Chamber Diameter   | 7500 lbf  |
| 8. Throttling Range           | Nominal Thrust, Minimal Thrust,<br>Control Complexity,<br>Added Components                   | 10:1  |
| 9. Space Based                | Vacuum Operation, No Purge<br>Gas Required   | No Change to Service Life   |
| 10. Man-Rated                 | Reliability #, Confidence Level,<br>Control Complexity,<br>Valve Design                      | Fail Operational, Fail Safe   |
| 11. Tank-Head Start           | Reliable Starting on Tank<br>Pressure  | NPSH: 15 ft-lbf/lbm for H <sub>2</sub><br>2 ft-lbf/lbm for O <sub>2</sub> |
| 12. Autogenous Pressurization | TPA Output Sufficient to Pressurize<br>Propellant Tanks above Cavitation<br>Minimum Pressure | TBD, Assume 25 psia, ± 5 psi  |

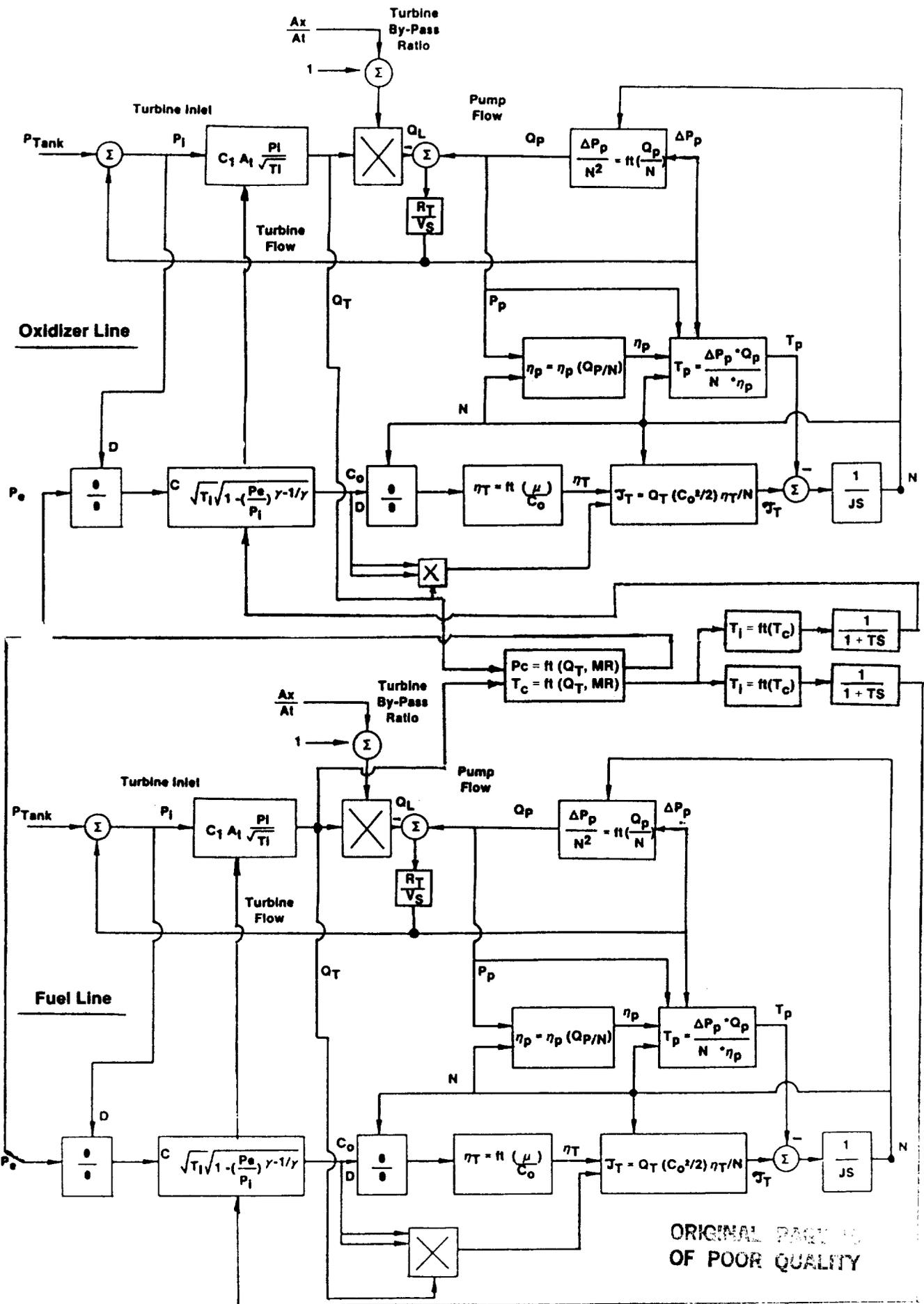


Figure 5. Engine Dynamic Simulation

### III, C, Model Building (cont.)

including a professional form for modeling large systems and for control and servo design. It allows for 999 blocks per model. Graphic support is required. This can be an IBM Color Graphics Board with color or NTS monochrome CRT, a Hercules High Resolution Graphics Board, an IBM EGA board in the default CGB mode, an IBM Dot Matrix PC and Epson Printers, or any other printer that will work with the IBM "Graphics" DOS command.

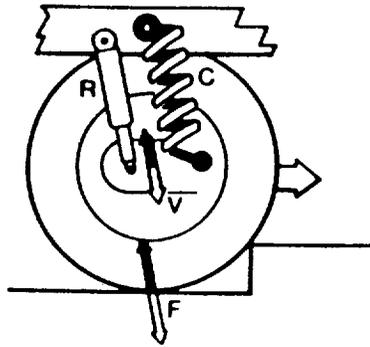
The TUTSIM<sup>®</sup> program is designed for the simulation of continuous dynamic systems. The key to use of the program is the preparation of the TUTSIM block diagrams. A problem is defined, mathematical models developed, a block model is composed based on the mathematical model, and the program presents results graphically. This is shown in Figure 6. Model parameters are easily entered and changed. Results are available in graphical display or numerical tables on the screen or as hard copy.

The program originated at Twente University of Technology in the Netherlands. Development has continued over 14 years with users now found worldwide. The core of TUTSIM is written in assembly language allowing for the fastest possible program execution. Non-linears, discontinuous, and special mathematical blocks add to its versatility. It is designed to be highly interactive allowing simulation interrupt with changes to parameters or output directives. TUTSIM can be called from other programs, modified, and re-run to optimize a parameter or for special analysis. The simulation algorithm allows the user to specify the delta time steps of the simulation. The entire model is "solved" at each delta time step. With well chosen time steps the solutions will closely approximate a continuous solution.

Program limitations are largely those of the personal computers in terms of speed and memory. The mathematical algorithms must be realistic, however, if the results are to be of any value. During the control modeling algorithm preparation was by far the most exacting and tedious task. Once the model was developed the program ran well.

Frequency domain analysis techniques are not included in this software. The two basic state equations of the model are derived from a summation of torques at the fuel and oxidizer turbopump assemblies. The angular acceleration

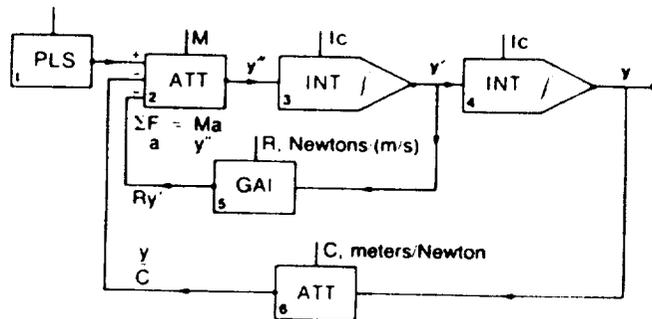
**Problem:**



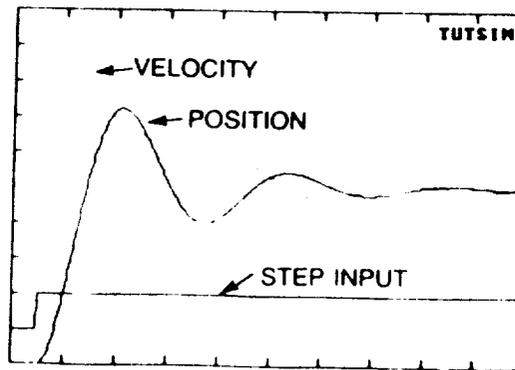
**Mathematical Model:**

$$y'' = \frac{1}{M} (F - Ry' - \frac{1}{C}y)$$

**Block Model:**



**Results:**



**Figure 6. Tutsim Program Application**

### III, C, Model Building (cont.)

of the turbopump (TPA) shaft is obtained from the difference in turbine and pump torques divided by the inertia of the TPA. Additional states in the model relate to the average pressure between the pump discharge and turbine inlet for both the oxidizer and fuel circuits. These particular state equations account for the capacitance of the system. Phase lags between the combustion chamber temperature and turbine inlet temperatures were also incorporated to simulate the effect of thermal capacitance. A list of symbols is given in Table III.

#### 2. Engine Component Modeling

The engine response during start-up and shut-down was obtained by combining the previously described turbopump dynamics model with a specified control configuration. Figure 7 illustrates the interrelationship between the two systems. The inputs are thrust and engine mixture ratio. These inputs are used to determine the required fuel and oxidizer flow rates. The turbine bypass valve schedule blocks represent table look-ups of the turbine bypass valve positions. Power balance runs at specified operating points through the throttling range were used to generate the bypass valve schedules (refer to Figure 8).

### D. START MODES/THROTTLING

#### 1. Tank Head Start

Requirements for the OTV engine include a capability for successful starts at tank pressures equivalent to the vapor pressure of hydrogen at  $37.8^{\circ}\text{R}$  and oxygen at  $162.7^{\circ}\text{R}$ . This is approximately 20 psia and 15 psia, respectively. On start the engine chamber is effectively at vacuum. The unaided maximum pressure differential available for start, then, is 15 psia which is set by the oxygen circuit. All line and component pressure drops must be accommodated within this pressure differential less a small combustion chamber pressure. This is a very small driving force for a rocket engine and poses a number of problems which are discussed below.

a. Pressure Fluctuations From Propellant Boiling. The engines are very likely at a temperature of 200 to  $300^{\circ}\text{R}$  higher than the entering propellant. Lines, turbopumps, and valves must be chilled to the same temperature as the propellant before pressure fluctuations from boiling propellants subside. During their

**Table III**  
**Symbols Used in the Simulation and Control Models**

**Symbols Used in Figure 11 and Table IV**

|              |                                    |
|--------------|------------------------------------|
| $Q_p$        | - Pump Flow                        |
| $N$          | - Pump Speed                       |
| $\Delta P_p$ | - Pump Pressure                    |
| $\tau_T$     | - Turbine Torque                   |
| $J$          | - Turbo-pump Inertia               |
| $X_y$        | - Valve Position of Pump Pre-Valve |
| $C, A_t$     | - Turbine Throat Equivalent Area   |
| $T_i$        | - Turbine Inlet Temperature        |
| $\eta_p$     | - Pump Efficiency                  |
| $\tau_p$     | - Pump Torque                      |
| $C_o$        | - Turbine Spouting Velocity        |
| $\eta_t$     | - Turbine Efficiency               |
|              | - Turbine Gas Specific Heat Ratio  |
| $P_c$        | - Chamber Pressure                 |
| $P_T$        | - Turbine Inlet Pressure           |

**Symbols Used in Figure 5**

|       |                            |
|-------|----------------------------|
| $P_e$ | - Turbine Exit Pressure    |
| $\mu$ | - Mean Turbine Blade Speed |
| MR    | - Engine Mixture Ratio     |
| $T_c$ | - Combustion Temperature   |

**Symbols Used in Figure 19**

|               |  |
|---------------|--|
| $U_{cone}(t)$ | - Commanded Actuator Positions   |
| $U(t)$        | - Actual Acuator Positions   |
| $U_m(t)$      | - Measured Actuator Positions (Includes Sensor Noise)                    |
| $Z(t)$        | - System Control Variables (e.g., Chamber Pressure, Turbine Speed, etc.) |
| $Z_m(t)$      | - Measured Control Variables   |
| $Z(t)$        | - Estimated Control Variables  |
| F             | - System Matrix  |
| G             | - Control Matrix   |
| H             | - Output Matrix  |
| D             | - Control Output Matrix  |

**Table III**  
**Symbols Used in the Simulation and Control Models (Cont)**

**Symbols Used in Figure 7**

|           |   |  |
|-----------|---|--|
| $M_f$     | - | Mass Flowrate of Fuel  |
| $M_o$     | - | Mass Flowrate of Oxidizer  |
| SV        | - | Sliding Valve Actuation (Proportional Valve)   |
| $X_v$     | - | Valve Position   |
| $(X_v)_R$ | - | Algorithm for Valve Positioning  |
| $(M_o)_R$ | - | Algorithm for Turbine Bypass Valve Positioning<br>Based on Oxidizer Mass Flowrate              |
| $(M_f)_R$ | - | Algorithm for Turbine Bypass Valve Positioning<br>Based on Fuel Mass Flowrate                  |
| N         | - | Feedback Signal from Oxidizer or Fuel Flow Used<br>for Closed Loop Control of Turbopumps or MR |
| D         | - | Feedback Signal from Fuel Flow Used for Closed<br>Loop Control of Mixture Ratio                |

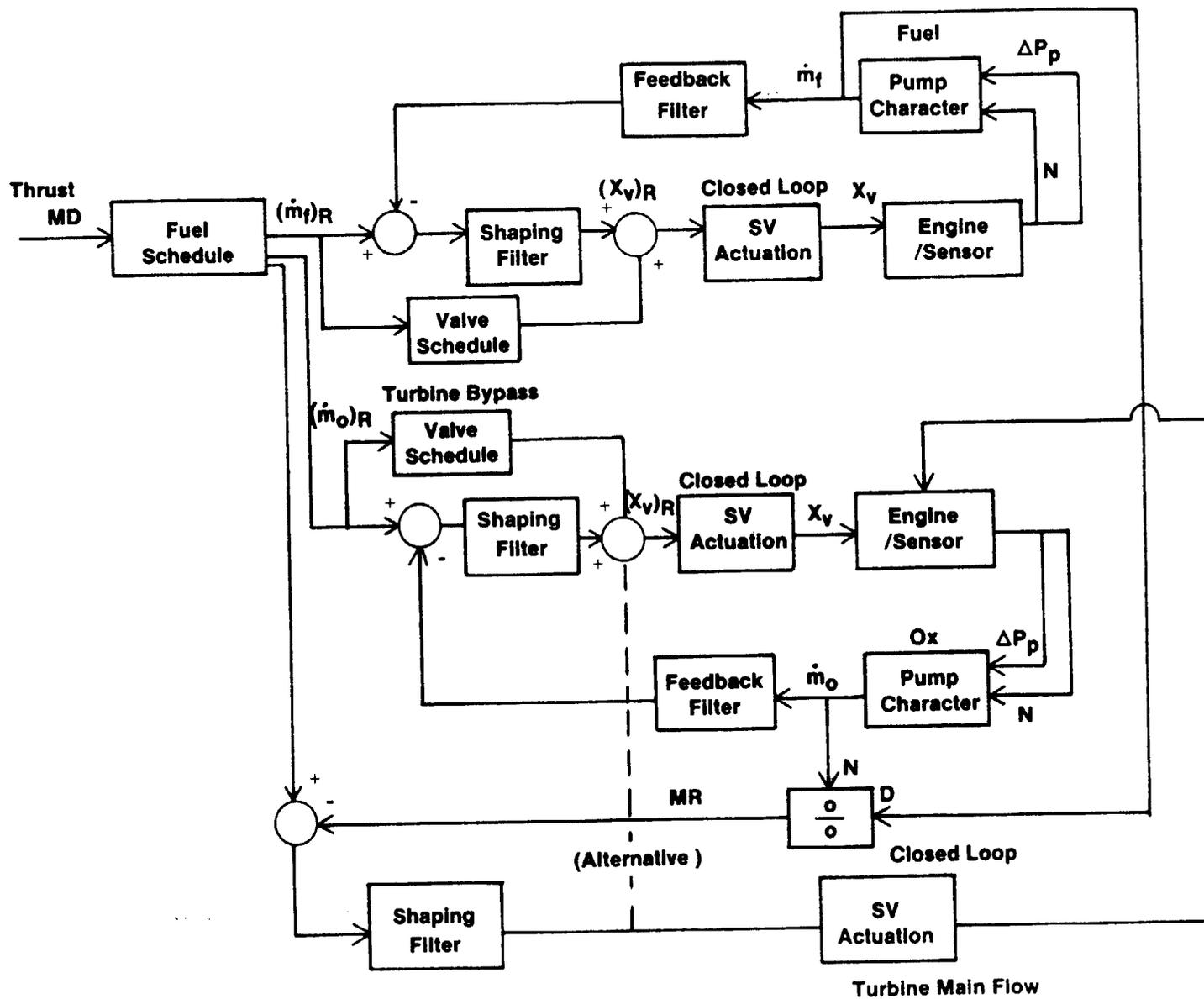


Figure 7. OTV Engine Control Configuration

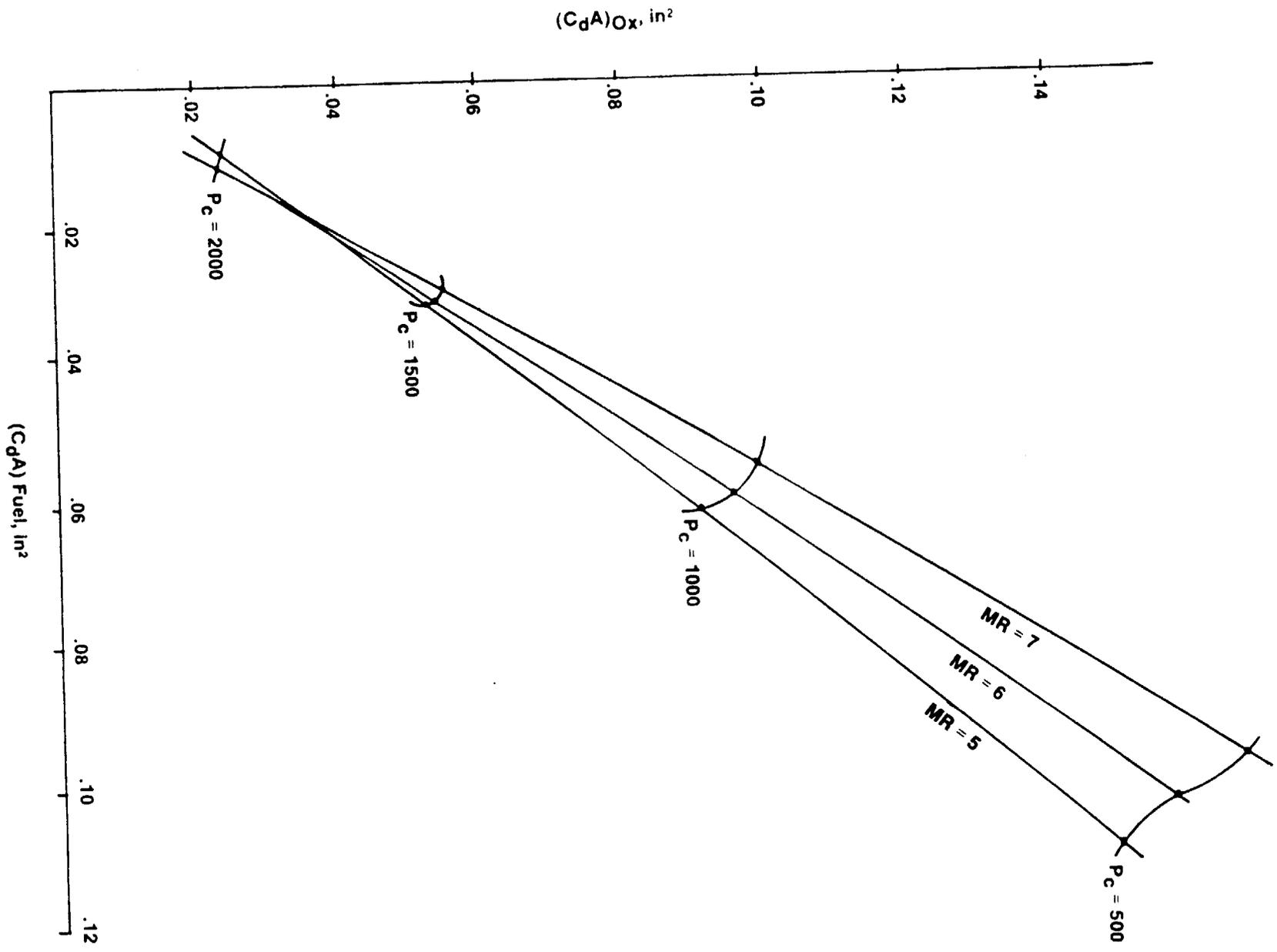


Figure 8. Turbine Bypass Valve Fuel Schedule

### III, D, Start Modes/Throttling (cont.)

chilldown period flowrates and pressures will be erratic within the flow circuits. A start at this time could be characterized by rapid chamber pressure fluctuations, uncontrollable mixture ratio changes, and intermittent loss of flame with subsequent re-ignitions and pressure spikes. Such operation must be avoided. One means is to dump a specific amount of the propellants through the engine without ignition. This can cost several hundred pounds of propellant to achieve chilldown. Another option is to recirculate the chilldown propellants back to the tanks through recirculation loops. For such loops to work, however, there must be a recirculation pump in each circuit. The obvious choice is a small electrically driven pump located near the propellant tank outlet. It does not have to have a flow capacity of more than 10% of the full-thrust flowrate. With such a pump chilldown will be rapidly completed with little or no loss in propellant. A bonus is rapid tank pressurization from the returning propellant due to heat pickup from the engine and propellant conversion to the gas phase.

The recirculation loop also requires added engine valves and lines to prevent the propellant from entering the injector during chilldown. The recirculation pumps can be continued in operation during the start to add a very useful boost in pressure to the engine. Once pumped idle is reached so that the autogenous pressurization system is operational they would be turned off.

A compromise start/chilldown procedure would use some initial propellant dumping with ignition started just as soon as sensor data indicated stable pressures and temperatures at the turbopumps.

b. Susceptibility to Combustion Transients. Combustion adds additional pressure fluctuations that, at low feed pressures, can be coupled to the feed system dynamics as a chug instability. This can be severe enough to damage equipment or couple with the vehicle dynamics to effect the whole vehicle as a pogo instability. The recourse is an immediate shutdown. Avoidance is best done by a rapid acceleration through the potential chug range to a stable engine operating point. We do not expect a prolonged period of operation at tank head idle for this engine; the available pressure drops are not high enough to preclude chug despite the expected smooth combustion with the proposed injector elements.

### III, D, Start Modes/Throttling (cont.)

c. Intermittent Combustion. The very low tank pressure requires a very low combustion chamber pressure for sustained operation. One concern is that the pressure spikes on lightoff may be high enough to stop propellant flow from one of the circuits. At that time the flame would extinguish only to be followed by another lightoff and pressure spike. This would be evident as a rapid popping that could be of increasing or decreasing magnitude. Continuous ignition during start would lessen the possibility of popping, but not entirely eliminate it. A better solution is to increase the system pressure to a value above the lightoff pressure spike.

d. Lack of Repeatability From Start-to-Start. A highly repeatable start transient is very useful in mission planning. With a tank head start there will be some unavoidable variability between starts. This can be avoided by compressing the time between lightoff and reaching a stable operating point. Prolonged operation at tank head idle would be characterized by large variations in delivered impulse for a given operating time. Such variations may be acceptable if they can be calculated in real-time.

e. Control Elements for Tank Head Start. A successful tank head start will require low pressure drop flow circuits. The engine schematic in Figure 4 shows a fuel idle valve. This valve opens on start to bypass the hydrogen turbopump circuit for a low pressure drop propellant route to the injector. As combustion stabilizes and more propellant flows through the TPA circuit the idle valve can be closed. Mixture ratio is controlled by the modulation of the variable position fuel idle valve.

f. Two Engine Thrust Vector Control. With side-by-side mounting of the two OTV engines the vehicle center of mass should be on a line between the two engines. Any difference in thrust between the two engines during start must be corrected by a gimbal movement of one or both engines or by firing of an attitude control thruster. Smoother operation will result if the start transient on both engines is very nearly identical and characterized by a rapid increase to a stable operating point.

### III, D, Start Modes/Throttling (cont.)

#### 2. Pumped Idle Mode

Pumped idle is defined as engine operation where the turbine power is sufficient to overcome the tare torque of the TPA. It represents minimum thrust operation with active turbomachinery. Up to that time the engine operating temperature and chamber pressure are relatively low since chamber pressure cannot exceed tank pressure minus system pressure drops. When the TPAs begin to rotate the propellant pressure in the engine system is no longer limited by tank pressure. As an operating point, pumped idle would be somewhat above the start rotation condition so that system fluctuations would not stall the pumps.

a. Control in Pumped Idle. At pumped idle the fuel idle valve is closed, and all propellant flows through the TPA circuits. The pressure operated back-pressure valves are functioning as high pressure drop flow restrictors (they never completely shut off flow and open completely when chamber pressure reaches 500 psia  $\pm$ 50 P psia). The hydrogen regenerator bypass and HGX bypass valves will be at 90% bypass (a margin of 10% is required for control authority) to control bulk temperature rise and chamber maximum wall temperature. The turbine bypass valves will be very nearly in the maximum bypass position but modulating for thrust and mixture ratio control.

The stability of this mode is highly dependent on control range (i.e., valve position range and flowmeter accuracy) and the tare torque differences between the two mechanically separate turbopumps. The two pumps must begin rotation at very nearly the same time; otherwise, there is a risk that the rapidly increasing output from the rotating pump will cause a mixture ratio change beyond the control of the bypass valves. This mode is very difficult to model accurately without actual pump tare torque and performance data.

#### 3. Engine Throttling.

The 10:1 throttling requirement is considered as a capability for fully controlled engine operation from 750 lbf thrust to 7500 lbs thrust at some mixture ratio between 5.0 and 7.0. There is no stated requirement for a specific rate-change in thrust although this is an important consideration in the controls design and there

### III, D, Start Modes/Throttling (cont.)

will be a practical rate limit. Relatively slow changes assure close control of mixture ratio.

a. Throttling Applications. There are three likely mission scenarios where throttling is needed or useful:

- 1) Landing a vehicle on the moon or Mars.
- 2) A gradual reduction in thrust during propellant depletion to control G-loads on payloads sensitive to acceleration forces.
- 3) Operation at lower than rated thrust to accomplish a precise orbit adjustment maneuver or rendezvous.

The first application will be the most demanding. If the engine control system can respond adequately to the moon lander mission needs, the other missions will be within the design envelope.

b. Throttling Control. Throttling is done primarily by commanding the oxygen turbine bypass valve to a specific position with the hydrogen bypass valve adjusting as needed to keep within the mixture ratio range. During throttle down operation the hydrogen regenerator bypass valve will allow more bypass due to the hydrogen bulk temperature rise through the chamber. The oxygen regenerator bypass valve would also increase the bypass based on the reduced power requirement of the ox TPA. The principal valves used during throttling, however, are the turbine bypass valves. Their action has been modeled and is given in Figure 9. The non-linear oxygen bypass prediction at low chamber pressures is due to the actuation of the back pressure valve. This non-linear operating regime will require a different control algorithm for reliable operation. One possibility is the use of TPA speed to calculate oxygen flow at chamber pressures below 600 psia to compensate for nonlinearities in flowmeters and the oxygen turbine bypass valve.

(Tti = 860 deg R)

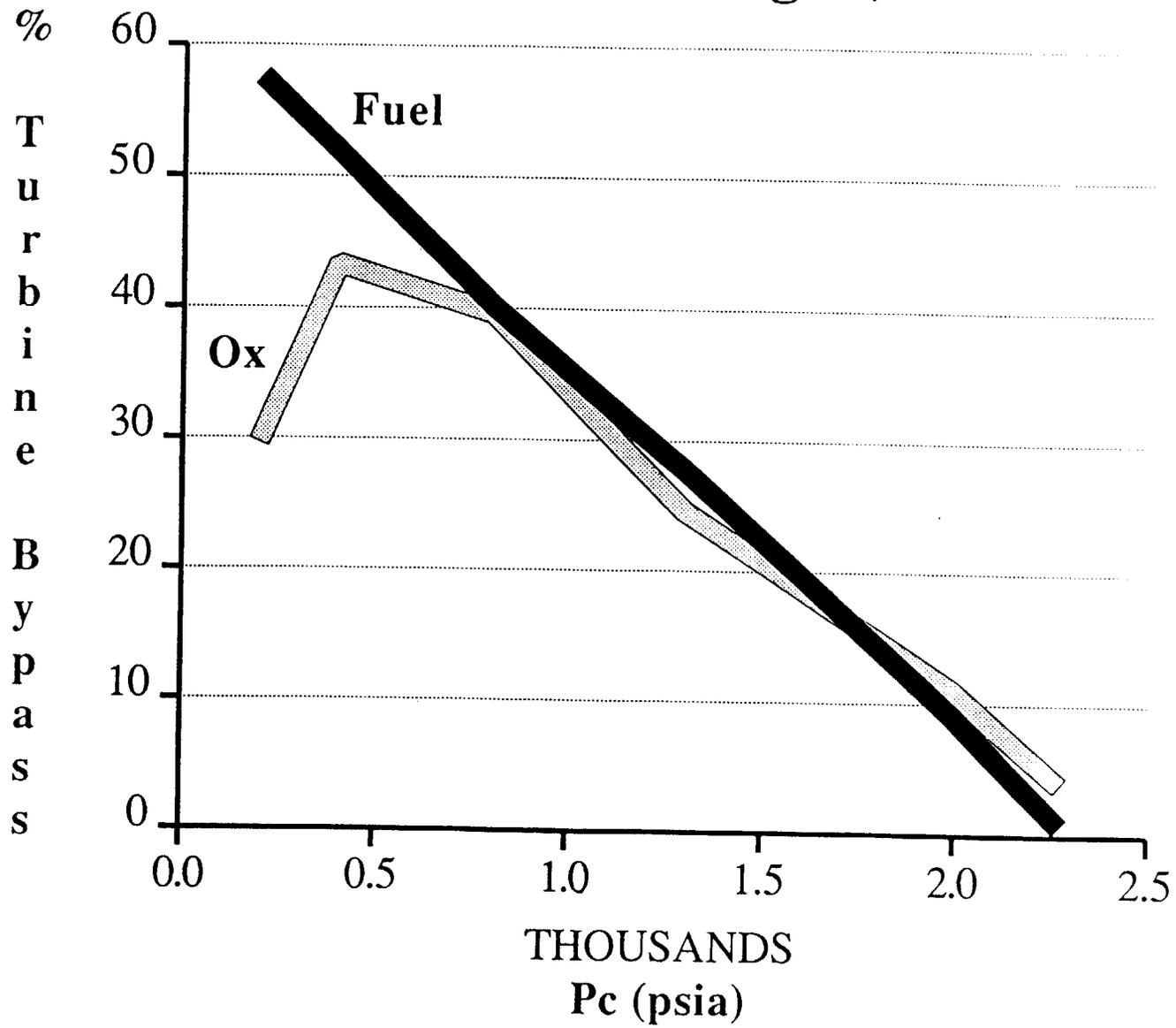


Figure 9. OTV Engine Turbine Bypass Percent vs. Chamber Pressure

#### IV. CONTROL LOGIC EVALUATION

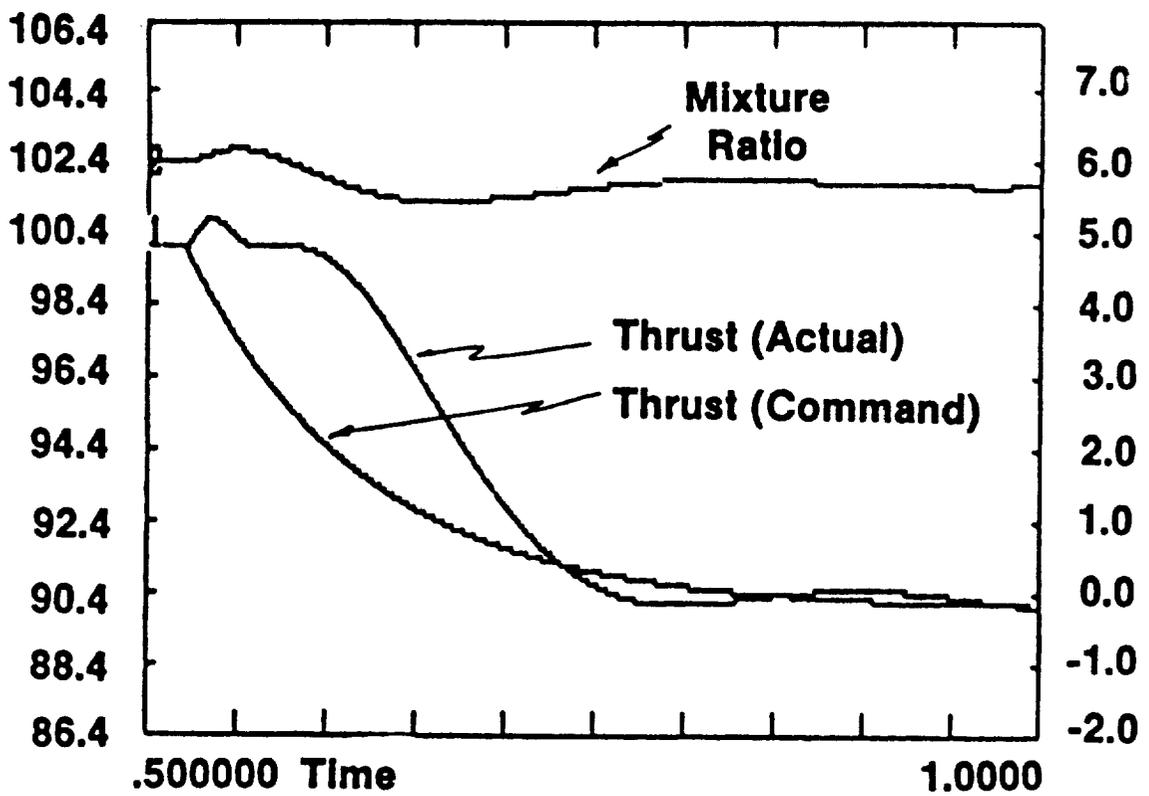
##### A. SYSTEM GAINS AND SHAPING FILTERS

For an uncompensated system there will always be an error between commanded thrust/mixture ratio and actual thrust/mixture ratio. This is due to such factors as general system disturbances and limitations in mathematical modeling used for valve scheduling calculations. To overcome this difficulty a feedback compensation configuration is utilized. The shaping filters and feedback gains are designed to provide adequate system stability and response times with no steady state error. For the model depicted in Figure 7 the shaping filters were assumed to be of the proportional plus integral type and the feedback filters were set as unity gains. The dynamic simulation results shown in Figures 10 and 11 were derived using the closed-loop model. From these results it can be seen that the system remains stable for changes in the command input signal. The simulation results, however, do not provide any information as to the overall stability of the system.

##### B. CONTROL SYSTEM STABILITY ANALYSIS

In order to determine the phase and gain margins, the system must first be linearized about an operating point. Once this has been accomplished, the overall linear transfer function between input and output is used to calculate the stability margins. In performing the linearization for this application, the fuel and oxidizer circuits were treated independently. Figure 12 shows the linearized propellant circuit equations. The partial derivative terms used in linearizing the torque expressions are given in Table IV. Analysis was started for determining system stability at the nominal thrust level. Additional work is still required to tune the system such that results from both the linear and nonlinear models match. Further analysis will be required to verify system stability through the entire throttling range. This can only be accomplished by identifying a number of discrete operating points spanning the dynamic range. For each operating point the partial derivative terms given in Table IV are evaluated. These values are then substituted into the linearized system transfer function. Model reduction techniques are used to simplify the transfer function so that frequency analysis methods may be utilized to determine the phase and gain margins.

**Y1: Thrust Command %**



**Figure 10. Predicted Response to 10% Throttle Down Command**

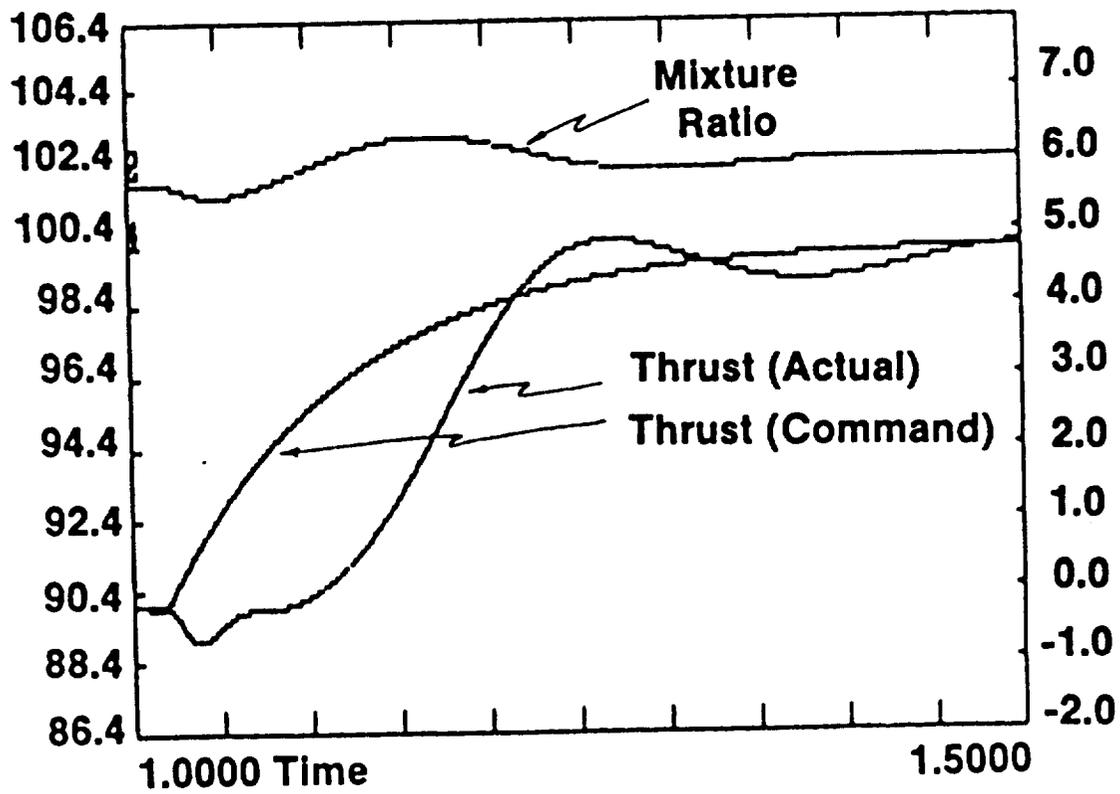
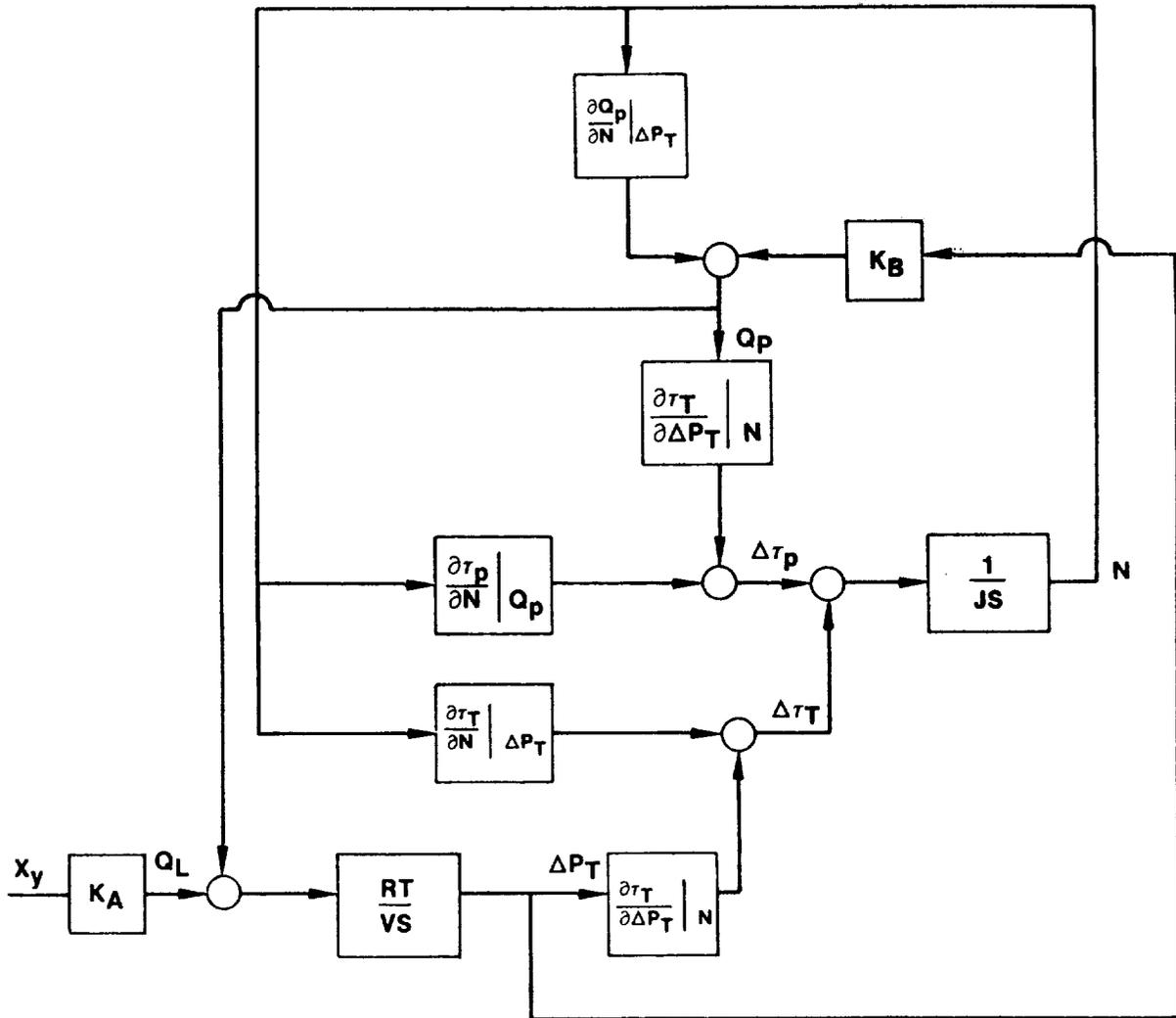


Figure 11. Predicted Response to 10% Throttle Up Command



Refer to Table III for Definition of Symbols

Figure 12. OTV Engine Linear Dynamic Model

**Table IV**  
**Parameters in the Engine Linear Dynamic Model**

$$\frac{\partial Q}{\partial X} = \frac{\frac{Q_p \sqrt{T_i}}{(1+x)^2 C_1 A_t}}{\frac{\sqrt{T_i}}{(1+x) C_1 A_t} - N \frac{\partial \text{ft} \left( \frac{\Delta P}{N^2} \right)}{\partial \left( \frac{Q_p}{N} \right)}}$$

$$\frac{\partial Q}{\partial N} = \frac{2 N \text{ft} \left( \frac{Q_p}{N} \right) - Q_p \frac{\partial \text{ft} \left( \frac{\Delta P}{N^2} \right)}{\partial \left( \frac{Q_p}{N} \right)}}{\frac{\sqrt{T_i}}{(1+x) C_1 A_t} - N \frac{\partial \text{ft} \left( \frac{\Delta P}{N^2} \right)}{\partial \left( \frac{Q_p}{N} \right)}}$$

$$\frac{\partial \tau_p}{\partial Q} = \frac{Q_p}{\eta_p} \frac{\partial \text{ft} \left( \frac{\Delta P}{N^2} \right)}{\partial \left( \frac{Q_p}{N} \right)} + \frac{\tau_p}{Q_p} - \frac{\tau_p}{N \eta_p} + \frac{\partial \eta_p}{\partial \left( \frac{Q_p}{N} \right)}$$

$$\frac{\partial \tau_p}{\partial N} = \frac{Q_p}{\eta_p} \left[ 2 \text{ft} \left( \frac{Q_p}{N} \right) - \frac{Q_p}{N} \frac{\partial \text{ft} \left( \frac{\Delta P}{N^2} \right)}{\partial \left( \frac{Q_p}{N} \right)} \right] - \frac{\tau_p}{N} + \frac{\tau_p}{\eta_p} \frac{Q_p}{N^2} \frac{\partial \eta_p}{\partial \left( \frac{Q_p}{N} \right)}$$

**Table IV**  
**Parameters in the Engine Linear Dynamic Model (Cont)**

$$\frac{\partial \Delta P_T}{\partial X} = \frac{\partial \text{ft} \left( \frac{\Delta P_p}{N^2} \right)}{\partial \left( \frac{Q_p}{N} \right)} \frac{N Q_p \sqrt{T_i}}{(1+x) \sqrt{T_i} - (1+x)^2 C_1 A + N \frac{\partial \text{ft} \left( \frac{\Delta P_p}{N^2} \right)}{\partial \left( \frac{Q_p}{N} \right)}}$$

$$\frac{\partial \Delta P_T}{\partial N} = 2 N \text{ft} \left( \frac{Q_p}{N} \right) + \frac{\partial \text{ft} \left( \frac{\Delta P_p}{N^2} \right)}{\partial \left( \frac{Q_p}{N} \right)} \left[ N \frac{\partial Q}{\partial N} - Q \right]$$

$$\frac{\partial \tau_T}{\partial \Delta P_p} = \frac{\tau_T}{\Delta P_p + P_T} + \left( \frac{2 \tau_T}{C_o} - \frac{Q_T \mu}{2N} \frac{\partial \eta_T}{\partial \left( \frac{\mu}{C_o} \right)} \right) *$$

$$\left[ \frac{C_2 \sqrt{T_i} P_c (\gamma-1)}{2 \gamma (P_T + \Delta P_p)^2 \left( \frac{P_c}{P_T + \Delta P_p} \right)^{\frac{1}{\gamma}} \sqrt{1 - \left( \frac{P_c}{P_T + \Delta P_p} \right)^{\frac{\gamma-1}{\gamma}}}} \right]$$

$$\frac{\partial \tau_T}{\partial N} = \frac{\pi D_p}{720} \frac{\tau_T}{\eta_T C_o} \frac{\partial \eta_T}{\partial \left( \frac{\mu}{C_o} \right)} - \frac{\tau_T}{N}$$

Refer to Table III for Definition of Symbols

## IV, Control Logic Evaluation (cont.)

### C. SYSTEM PERFORMANCE

A basis of comparison is necessary in analyzing and designing control systems. Typical test signal inputs are step functions, ramp functions and acceleration or parabolic functions. By using a standard input which is representative of an actual input, a measure of system transient response can be made. For the OTV system the transient response of a unit step for thrust could be used to obtain the response time, percent overshoot and steady-state error. If the results of the analysis are unacceptable, for example the response time is too slow, the shaping filters and gains would be adjusted until satisfactory results are obtained. The response times of the preliminary model, at nominal thrust conditions (refer to Figures 10 and 11) are shown to be satisfactory. Simulations at many operating points need to be performed in order to verify the transient response over the throttling range. It is possible that a variable gain schedule may be required to achieve both adequate stability and response over this range.

### D. CONTROL MODEL UPDATES

Future model updates that will add to the dynamic model fidelity include: 1) modeling of hydrogen regenerator, 2) a more accurate model of the combustion chamber heat transfer process, 3) additional line and cooling jacket capacitance equations, 4) the use of actual valve area characteristic curves as a function of stroke and valve actuator models and, (5) modeling of the LOX/GH<sub>2</sub> heat exchanger. It should be noted that the linearization process described previously only applies to single input - single output systems. In actuality the OTV system is a multiple input - multiple output system; i.e., thrust and mixture ratio; however, since the desired mixture ratio was assumed to be constant, the analysis is pertinent. A state-space representation is commonly employed for multi-variable feedback systems. This treatment provides analytic tools for determining the observability and controllability of the state variables, accounts for cross coupling effects, and can be used for determining optimum variable feedback gains over the range of operation. Software such as MATRIX-X, which is an integrated modeling, simulation, and control system package is available for this analysis. The control design module of MATRIX-X includes all linear system analysis functions including interactive classical and modern control design, conversion between model forms, and compu-

tation of time and frequency responses. Non-linear models can be created and simulated interactively. There are six different integration algorithms available for simulating continuous, hybrid (mixed continuous/discrete), and even multi-rate digital systems. A system identification module is also provided which enables the user to transform test data into accurate system models. It is available to facilitate modern controls analysis. One standard feature allows the user to transfer from the Laplace domain, used in classical controls, to the state-space representation of modern controls. The computing ability of this software will be very useful for future work if optimization or multiple feedback variable modeling become important design considerations.

#### E. CONTROL CONFIGURATION

The control configuration shown in Figure 7 is an approximation of the plant dynamics and feedback compensation for the system. A more detailed control logic is presented in Figure 13. In this concept the two basic feedback parameters are chamber pressure and oxidizer flow rate. The thrust and mixture ratio commands are rate-limited to avoid the possibility of severe system oscillations and thermal spikes or TPA stall. The commanded chamber pressure is calculated based on thrust and mixture ratio and fed forward to the feedback summing junctions and fuel and oxidizer turbine bypass valve schedules. Error signals between commanded chamber pressure and actual chamber pressure are fed through proportional plus integral controllers. These filtered signals are then summed with the table look-up stroke commands. At this point the total valve stroke signal actuates the valve to provide the desired compensation. The valve schedules are a multiple set of curves that give valve stroke as a function of engine chamber pressure and mixture ratio. Their basic function is to provide reasonable initial valve stroke positions so that controller modulation is kept to a minimum. Mixture ratio is controlled in the same manner as chamber pressure, with the difference that only the hydrogen turbine bypass valve is modulated for a steady position of the oxygen turbine bypass valve. The time base start and shutdown valve schedules for the fuel circuit are shown in Figures 14 and 15. Note that all the valves, with the exception of the turbine bypass valves, the regenerator bypass valves, the LOX/GH<sub>2</sub> heat exchanger valve and the fuel idle valve, are on-off valves. The on-off valve schedules are predetermined using dynamic simulation runs in order to achieve desirable start and shutdown characteristics. Included in the control logic is compensation for turbine inlet tem

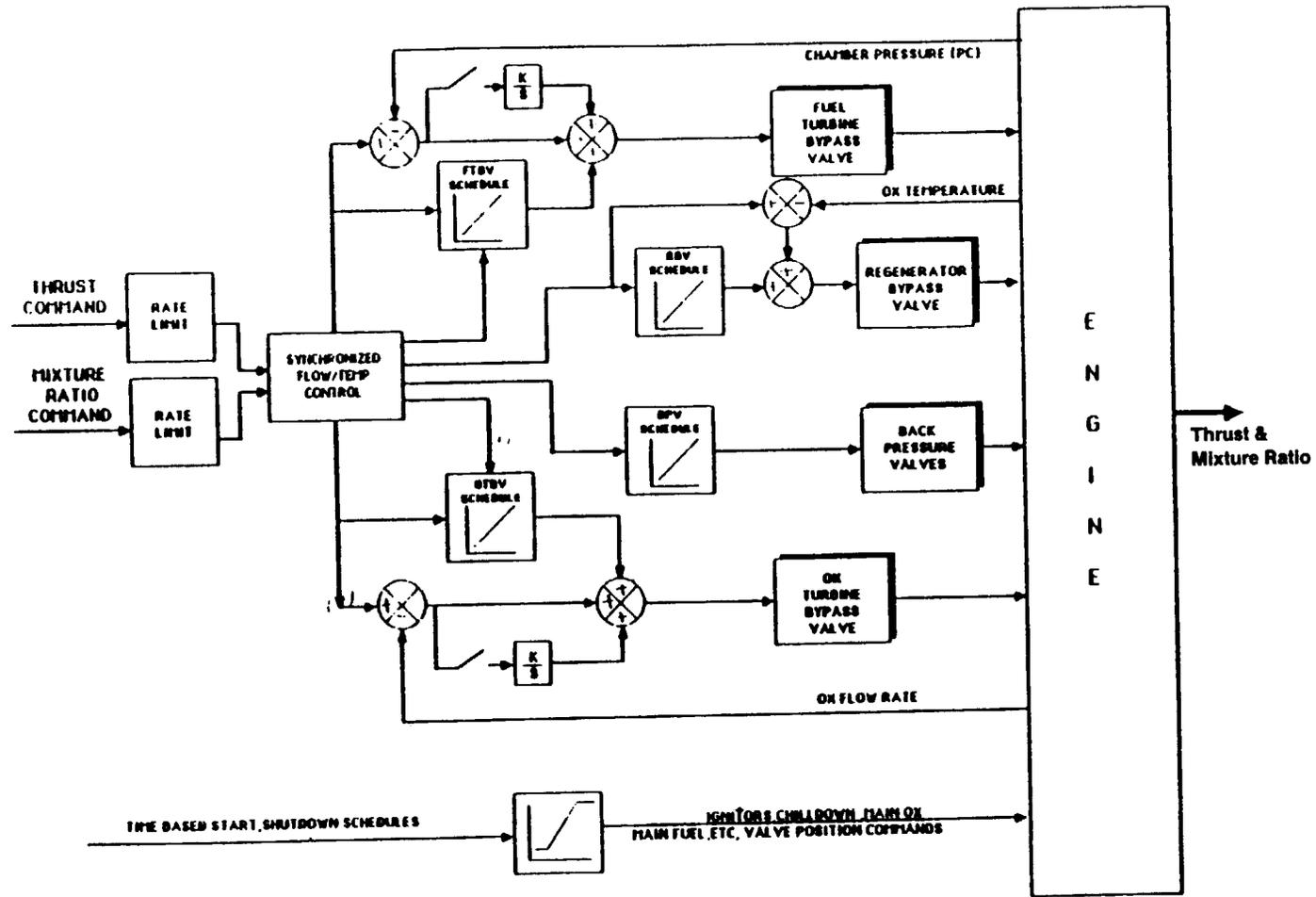


Figure 13. Selected Control Logic Provides Stable Engine Operation

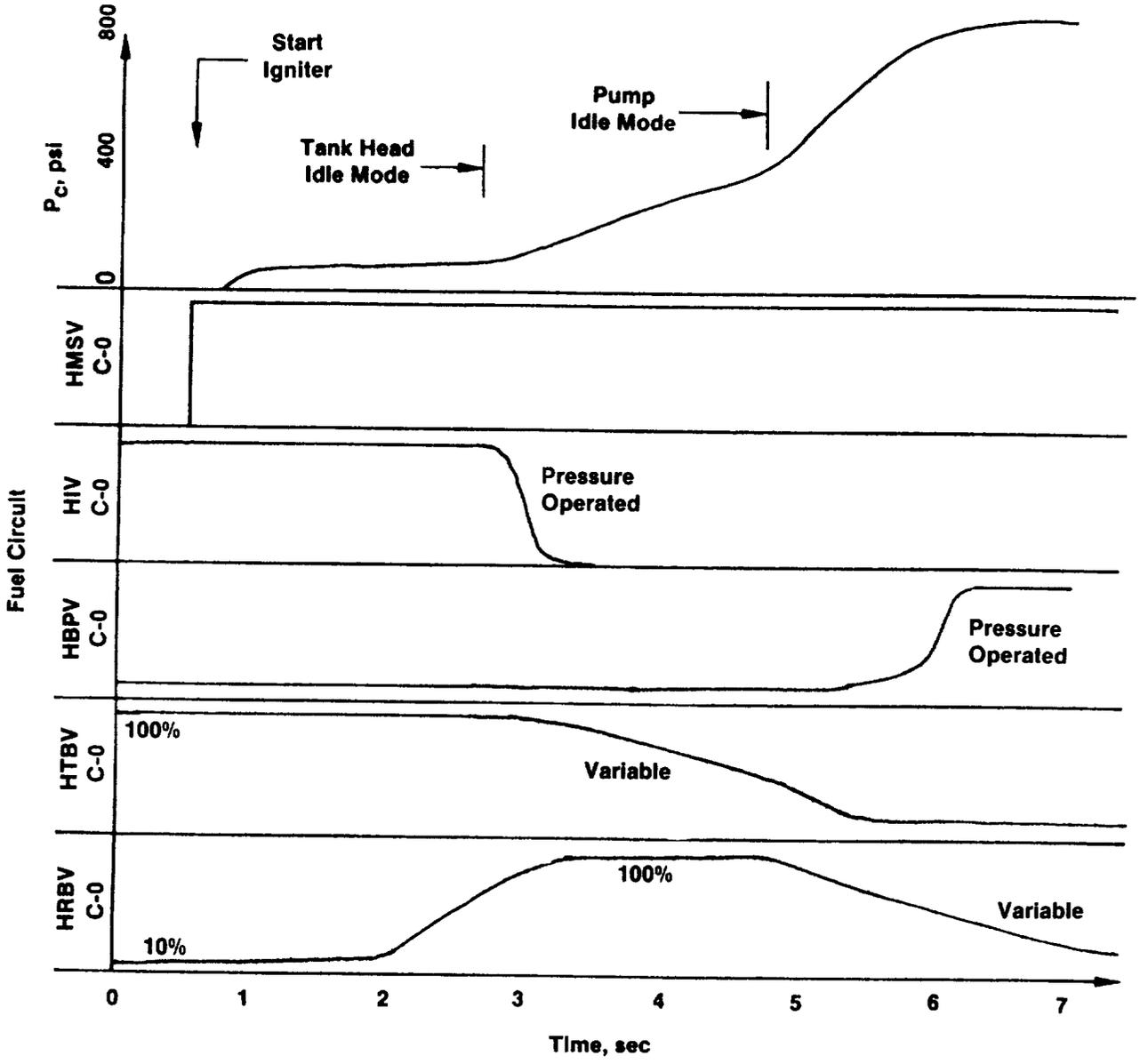


Figure 14. OTV Engine Start-up Sequence

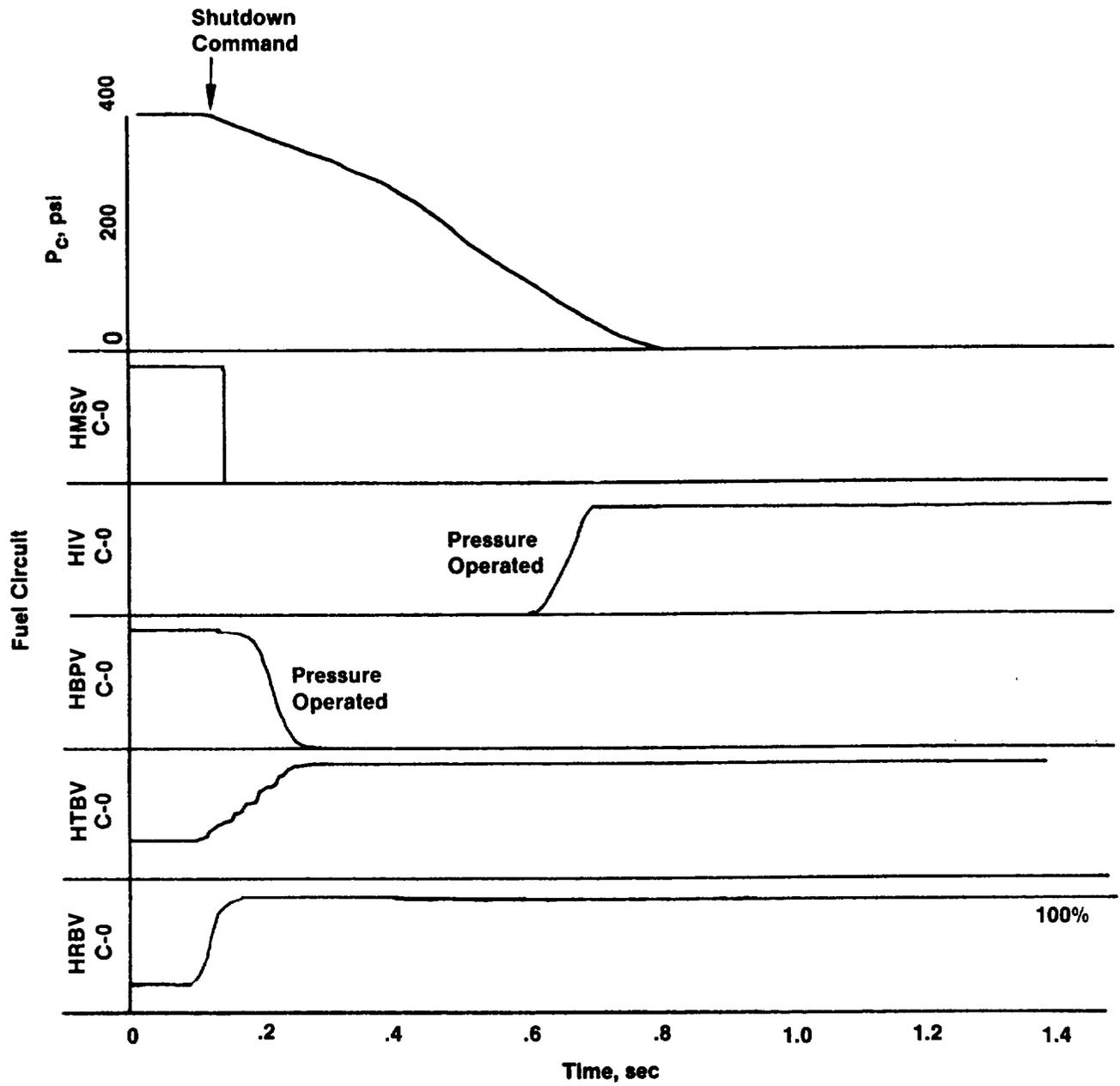


Figure 15. OTV Engine Shut-down

#### IV, E, Control Configuration (cont.)

perature variations by use of regenerator bypass valves. The back pressure valves impose a high pressure drop during low chamber pressure engine operation so that the liquid oxygen converts to a high pressure gas under conditions well above the two phase dome. This is necessary to prevent two-phase flow into the injector.

## V. CONTROL ELEMENT REQUIREMENTS

There are four basic control elements: 1) the various flow control valves, 2) those sensors specifically identified for engine controller use, 3) the controller hardware, and 4) the control software/firmware. In addition, supplementary information can be gained from health management sensor data if suitably processed. For a piloted version of the OTV, commands from the pilot's station can direct controller operation. An unmanned OTV would have vehicle guidance computer and telemetry inputs that would direct controller (and engine) operation.

### A. CONTROL VALVE REQUIREMENTS

The basic control valve configuration is illustrated in Figure 4b. Of the 12 valves depicted, five are modulation valves and seven are on-off valves. Table V is a list of the individual valve functions. The valve operating environments are based on engine operating range extremes. Single page specifications for each valve are given in Appendix B.

### B. CONTROL SENSOR REQUIREMENTS

Table VI gives a listing of the specified engine sensors, their purpose, operating ranges and types. Appendix B includes a complete listing of sensors with a summary of the ranges and operating characteristics. The sensors given in Table VI are necessary elements of the control system and serve as a logical starting point for developing sensor requirements for health management applications.

### C. ENGINE CONTROLLER REQUIREMENTS

ATC has baselined the use of a controller developed under an IR&D program. This engine controller may be readily adapted to handle all of the OTV engine control requirements. Important features of the IR&D controller are: the use of hybrid microelectronics which are inherently small, light and rugged; dual channel redundancy architecture to eliminate single point failures; low power requirements; small size and low weight. The controller requirements of the OTV engine, which were compiled on the basis of the selected control logic, sensors and valves, is given in Table VII. The corresponding IR&D interface data are found in Table VIII. By comparing the information given in Tables VII and VIII, it is appar-

TABLE V  
OTV ENGINE CONTROL VALVES

| <u>VALVE NAME</u>                          | <u>ABBREVIATION</u> | <u>FUNCTION</u>                       | <u>FLUID</u>          | <u>OPERATING ENVIRONMENT</u>              | <u>TYPE</u>  |
|--|---------------------|---------------------------------------|-----------------------|---|--|
| Oxygen Main Shutoff                        | OMSV                | Propellant Isolation                  | Liquid O <sub>2</sub> | 0 to 45 psia<br>-310 to -298°F            | PO were Open/<br>Solenoid Latch/<br>Spring Loaded<br>Closed    |
| Hydrogen Main Shutoff                      | HMSV                | Propellant Isolation                  | Liquid H <sub>2</sub> | 0 to 45 psia<br>0 to 40°R                 | Powered Open/<br>Solenoid<br>Latch/<br>Spring Loaded<br>Closed |
| Oxygen Turbine Bypass                      | OTBV                | Mixture Ratio Control                 | Hot GOX               | 0 to 5000 psia<br>0 to 500°F              | Servo/Pintle<br>Fails in Place                                 |
| Hydrogen Turbine Bypass                    | HTBV                | Thrust Control                        | Hot GH <sub>2</sub>   | 0 to 6000 psia<br>0 to 1000°F             | Servo/Pintle<br>Fails in Place                                 |
| 42<br>Hydrogen Regenerator<br>Bypass Valve | HRBV                | Fuel Circuit<br>Temperature Control   | Cold GH <sub>2</sub>  | 0 to 5600 psia<br>-340 to -420°F          | Modulating<br>Poppet<br>Fails in Place                         |
| Heat Exchanger Bypass<br>Valve             | HEBV                | Oxygen Circuit<br>Temperature Control | Hot GH <sub>2</sub>   | 0 to 2700 psia<br>0 to 800°F              | Modulating<br>Poppet,<br>Fails in Place                        |
| Hydrogen Idle Valve**                      | HIV                 | Idle Operation<br>MR Control          | GH <sub>2</sub>       | 80 psid, 0 to 6000<br>psia -360 to 540°F  | Modulating<br>poppet, fails<br>closed**                        |
| Oxygen Back Pressure                       | OBPV                | Heat Exchanger<br>Pressure Control    | Warm GO <sub>2</sub>  | 200 psid, 0 to 5000<br>psia -250 to 400°F | Open/Partial*<br>Closure                                       |
| Hydrogen Back Pressure                     | HBPV                | Pressure Balance<br>During Start      | Warm GH <sub>2</sub>  | 200 psid, 0 to 6000<br>psia -360 to 600°F | Open/Partial*<br>Closure                                       |

TABLE V (cont.)  
OTV ENGINE CONTROL VALVES

| <u>VALVE NAME</u>               | <u>ABBREVIATION</u> | <u>FUNCTION</u>                            | <u>OPERATING FLUID</u>            | <u>ENVIRONMENT</u>              | <u>TYPE</u>                              |
|---------------------------------|---------------------|--|-----------------------------------|---------------------------------|--|
| Igniter Flow Control(2)***      | OICV,<br>HICV       | Igniter Propellant<br>Flow Control         | GH <sub>2</sub> , GO <sub>2</sub> | 0 to 6000 psia<br>-400 to 100°F | Powered/Open<br>Spring Loaded<br>Closed  |
| Oxygen Tank<br>Pressurization   | OTPV                | Oxygen Tank<br>Pressurization<br>Control   | Warm GO <sub>2</sub>              | 0 to 3000 psia<br>0 to 400°F    | Powered Open/<br>Spring Loaded<br>Closed |
| Hydrogen Tank<br>Pressurization | HTPV                | Hydrogen Tank<br>Pressurization<br>Control | Warm GH <sub>2</sub>              | 0 to 5000 psia<br>0 to 800°F    | Powered Open/<br>Spring Loaded<br>Closed |

As of April 1988.

\* Pressure operated. No electrical actuation devices. These valves are used to raise circuit pressure high enough to avoid the two-phase dome for the oxygen heating and the consequent film boiling in the heat exchanger.

\*\* A pressure operated valve is an option.

43 \*\*\* Not shown on engine schematic.

**Table VI**  
**Engine Sensors for Baseline Control Logic**

|             | MEASUREMENT                 | PURPOSE             | OP. RANGE      | TYPE  | FLUID       |
|-------------|-----------------------------|---------------------|----------------|---|-------------|
| PRESSURE    | 1 CHAMBER                   | THRUST CONTROL      | 0 TO 2500 PSIA | STRAIN GAGE (TEMP. COMP.)                       | HOT G02+GH2 |
|             | 2 OX. PUMP DISCHARGE        | OX. DENSITY CALC.   | 0 TO 4500 PSIA | STRAIN GAGE (TEMP. COMP.)                       | L02         |
|             | 3 FUEL PUMP DISCHARGE       | FUEL DENSITY CALC.  | 0 TO 4500 PSIA | STRAIN GAGE (TEMP. COMP.)                       | LH2         |
| TEMPERATURE | 1 OX. PUMP DISCHARGE        | OX. DENSITY CALC.   | -310 TO -260 F | RESISTANCE TEMPERATURE DETECTOR (RTD)           | L02         |
|             | 2 FUEL PUMP DISCHARGE       | FUEL DENSITY CALC.  | -420 TO -340 F | RESISTANCE TEMPERATURE DETECTOR (RTD)           | LH2         |
|             | 3 OX. TURB. INLET           | TEMP. LIMIT CONTROL | 0 TO 500 F     | RESISTANCE TEMPERATURE DETECTOR (RTD)           | HOT G02     |
|             | 4 FUEL TURB. INLET          | TEMP. LIMIT CONTROL | 50 TO 400 F    | RESISTANCE TEMPERATURE DETECTOR (RTD)           | HOT GH1     |
| FLOW RATE   | 1 OX. PUMP DISCHARGE        | OX. WDOT CALC.      | 0 TO 0.25 CFS  | ULTRASONIC FLOW METER                           | L02         |
|             | 2 FUEL PUMP DISCHARGE       | FUEL WDOT CALC.     | 0 TO 0.68 CFS  | ULTRASONIC FLOW METER                           | LH2         |
| VALVE POSN  | 1 OX. TURBINE BYPASS VALVE  | SERVO VALVE CONTROL | TBD IN         | LINEAR VARIABLE DIFFERENTIAL TRANSFORMER (LVDT) | TBD         |
|             | 2 FUEL TURBINE BYPASS VALVE | SERVO VALVE CONTROL | TBD IN         | LINEAR VARIABLE DIFFERENTIAL TRANSFORMER (LVDT) | TBD         |
|             | 3 OX. BLE VALVE             | SERVO VALVE CONTROL | TBD IN         | LINEAR VARIABLE DIFFERENTIAL TRANSFORMER (LVDT) | TBD         |

**Table VII**  
**Selected Control Logic, Sensors and Valves Drive the Controller Requirements**

|   |                                 |
|---|---------------------------------|
| <b>Vehicle to Controller Interfaces:</b><br>Avionics Power Bus<br>Utility Power Bus<br>Serial Command Channels<br>Power-up Command Channels<br>Discrete Status Channels   | 1<br>1*<br>1*<br>1<br>1*        |
| <b>Controller-to-Engine Electrical Interface:</b><br>On/Off Valves<br>Throttling Valves (Analog Commands)<br>Pressure Sensors<br>Temperature Sensors<br>Speed** Sensors or Vortex Shedding Flowmeters<br>Valve Position (Analog Inputs)<br>Regulated Avionics | 7<br>3<br>3<br>4<br>2<br>3<br>1 |
| <b>Ground Support Equipment Interface:</b><br>Serial I/O<br>Discrete Inputs<br>Analog Outputs (Test Points)   | 1<br>1<br>4                     |

\*Optional

\*\*At Low Speeds both TPA Speed and Flowmeter Readings must be used to Assure Adequate Mixture Ratio Control.

**Note: Quantities Listed are Per Channel**

TABLE VIII

IR&D ENGINE CONTROLLER - ENGINE INTERFACES

| <u>Function</u>                          | <u>Quantity</u> |
|--|-----------------|
| Solenoid Valves/Devices                  | 12              |
| Pressure Sensors                         | 8               |
| Temperature Sensors - TC                 | 2               |
| Temperatures Sensors - RTD               | 2               |
| Turbine Speed Sensors                    | 2               |
| Valve Position Sensors                   | 1               |
| Analog Commands                          | 2               |
| Discrete Commands                        | 8               |
| Analog Inputs                            | 12              |
| Vortex Shedding Flowmeters (2 in series) | 4               |

(Quantities Listed On a Per Channel Basis)

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TABLE VIII (cont.)

IR&D ENGINE CONTROLLER - VEHICLE INTERFACES

| <u>Function</u>           | <u>Quantity</u> |
|---------------------------|-----------------|
| 28 VDC Avionics Power     | 1               |
| 28 VDC Utility Power      | 1               |
| Power-Up Command          | 1               |
| Discrete Command Channels | 3               |
| Serial Command Channels   | 1               |
| Serial Data Channels      | 1               |

(Quantities Listed On A Per Channel Basis)

V, C, Engine Controller Requirements (cont.)

ent that the IR&D controller could be used on the OTV engine with minimal modifications.

The IR&D controller has a 128K word memory capacity; approximately 34K of this memory is required for engine control. Because the IR&D controller has a generous amount of surplus memory space, its application to the OTV program should not impose any limitations to control capabilities.

Additional considerations dictated by multi-engine control remain to be examined. For example, start transient coordination, engine-out coordination compensation and gimbaling are issues that will effect the final engine controller requirements for a multi-engine system.

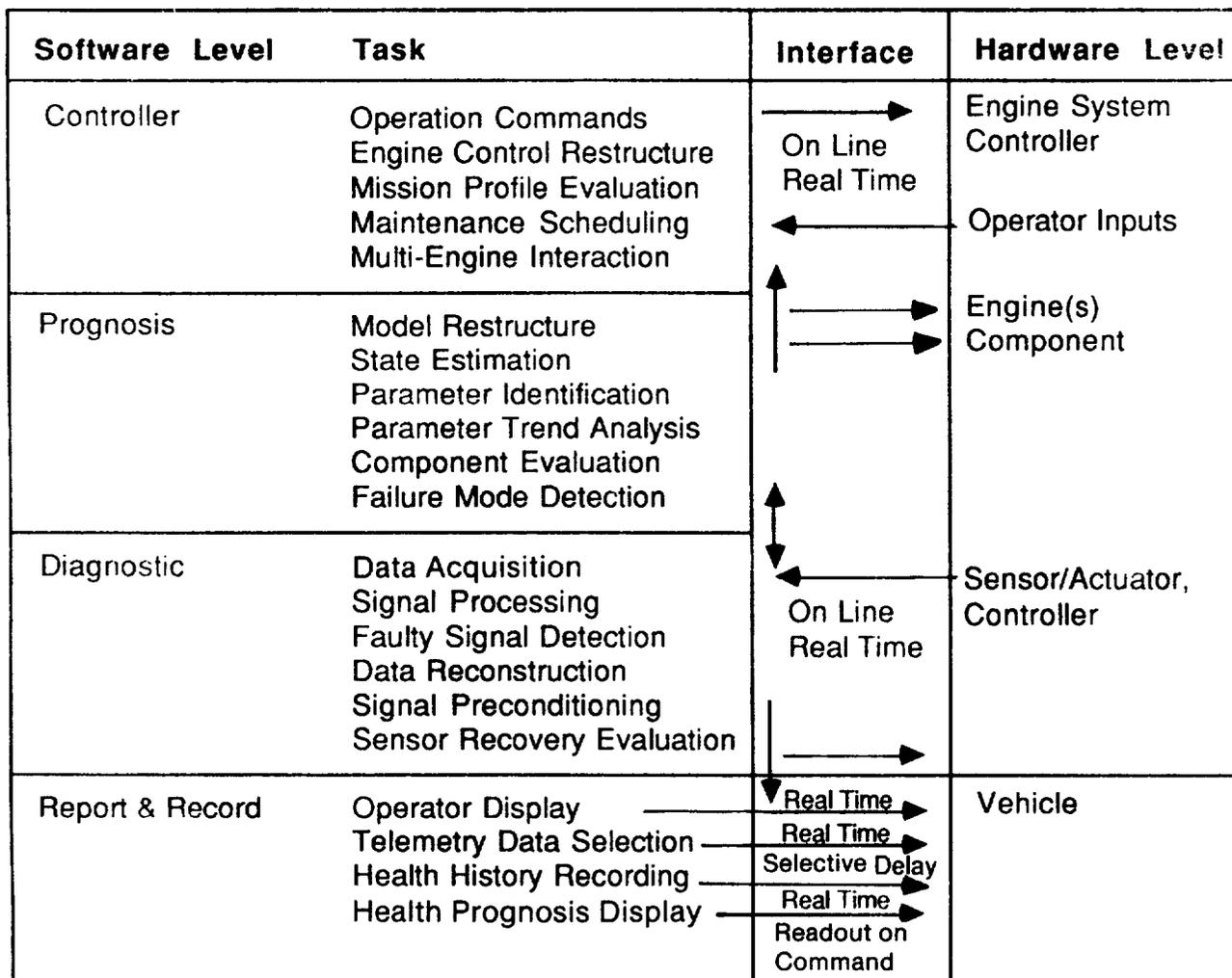
## VI. HEALTH MANAGEMENT

### A. HEALTH MANAGEMENT SYSTEM DESCRIPTION

The rationale for a health management system is to eliminate the high cost of losing a mission because of engine failure, and to reduce the high cost of returning space based engines to earth for servicing. It is a collection of methods for improving engine system reliability, performance and cost effectiveness through early fault detection and preventive maintenance. Functions provided by the health management system include: pre-flight engine check-out, detection of out-of-limit conditions, fail-operational/fail-safe fault tolerance for all flight critical functions, recognition of sudden shifts in performance, trend data collection and storage, off-line extrapolation of performance trends to plan scheduled maintenance, and modular fault isolation to permit more cost effective maintenance. The health management system functions continuously during engine operation and during pre- and post-checkout. At pre-flight testing it will evaluate the health of the engine sensors. After the flight it will process collected data and signal commands for required maintenance. The health management system will sequentially collect signals from sensors, process the signal data for efficient use, diagnose the engine component behaviors and evaluate actual engine system performance against commanded performance during the flight mode. Figure 16 illustrates the health management system flow paths.

### B. ENGINE FAILURE MODES

A compilation of internal and component OTV engine failure modes is presented in Table IX. The health management system is responsible for detecting anomalies during engine operation caused by any of these failure modes either by direct sensor measurement or computational algorithms. The hierarchical structure of the health management system is shown in Figure 17. The red-line sensors are of primary importance for engine prognosis. If the red-line limits are exceeded during engine operation, shut-down commands are issued to the controller processor in order to avoid a catastrophic failure. Many of the red-line parameters are not measurable quantities; however, state estimation algorithms can be used to ascertain the unknown values from the measured data. The pink-line sensors are at a lower level in the health management architecture. The information provided by these sensors is compared with the established pink-line limits. Engine operation



Arrows indicate the direction of data flow or commands. For instance, vehicle operator (pilots or a control center remotely located but in radio contact with the vehicle) can make inputs through an interface to the engine controller. The HM system also channels information to the operator display as either real time data, selective data, analyzed data with a time delay, or data commanded by the operator.

Figure 16. Health Management System Configuration

TABLE IX

OTV ENGINE FAILURE MODES

| <u>Component</u>      | <u>Failure Modes</u>   |
|-----------------------|--|
| TPA                   | Pressure Loss in Hydrostatic Bearing<br>Turbine Blade Chipping or Cracking<br>Oxygen Turbine Fire due to Rubbing<br>Nozzle Erosion Crack<br>Corrosion or Fatigue Failure<br>Pump Cavitation Damage<br>Seal Rupture |
| TCA                   | Blanching<br>Cooling Channel Contamination<br>Injector Contamination<br>Chamber Fatigue Cracking<br>Connection Leak<br>Overheating/Undercoolng   |
| Line                  | Connection Leak<br>Line Cracks   |
| Valves                | Open Circuit (Electrical)<br>Sensor Malfunction (No Position<br>Indictation)<br>Electrical Short   |
| Engine Sensor         | Malfunction (Change in Bias, Failure)<br>Interface Break-Down  |
| Igniter<br>Controller | Igniter Valve Failure<br>Spark Plug Failure<br>Open Circuit/Short<br>Module Break-Down<br>Interface Break-Down<br>Cooling System Failure<br>Algorithm Defect   |

- EACH LAYER OPTIMIZED FOR ITS OWN FUNCTION
- CONTROL ISOLATED FROM HEALTH MANAGEMENT FAILURE
- MODULAR STRUCTURE AIDS DEVELOPMENT

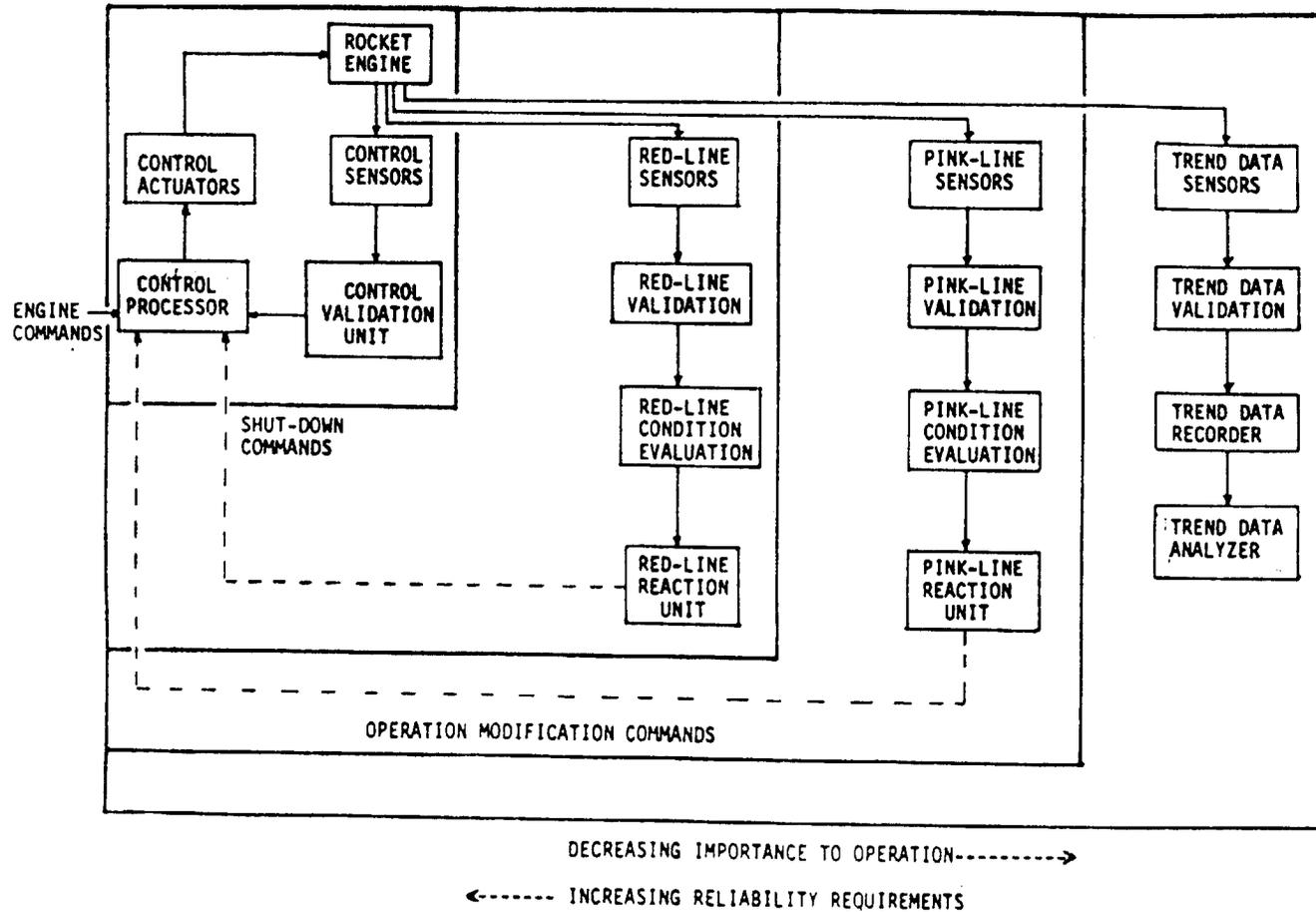


Figure 17. Layered HM/C System Architecture

## VI, B, Engine Failure Modes (cont.)

modification commands are sent to the control processor if any of the pink-line limits are exceeded. Prompt preventive measures during engine operation are needed to avoid more serious consequences. The generalized detection algorithm flow diagram is depicted in Figure 18.

Component or engine failures can be due to external causes as well as faults within the engine system. The space environment includes high radiation levels that can destroy electronic parts and weaken structural materials. A comprehensive health management system should include radiation dose recorders with an alerting system when critical levels are approached. Also there is an ever present hazard from micrometeorites and space debris that could be responsible for a variety of failures. The health management and control system must be able to effect a timely engine shutdown after such an encounter. The aerobraking maneuver also poses some hazards. Failure of an engine cover door would place the engine system in a high temperature, highly fluctuating aerodynamic load environment. The likely failure would be in the extendible exit cone, but high temperatures could destroy control elements. The health monitor system would have to assess damage prior to any engine start attempt. The multiple possibilities for electrical failure require mechanical backup such as valves that are spring loaded closed on power failure. These are the most obvious externally caused failures, but there are others. The health management system should be designed to safely handle all of them.

### C. DESIGNING FOR SYSTEM RELIABILITY

Any complex data system must be designed to accommodate normal operating fluctuations and measurement error. Since noise and general system disturbances are random processes, statistical methods are used to filter all measured data. The reliability of the digital electronic control system can be improved by incorporating control component redundancy at the sensors, the actuators, or the controls computer itself.

The two types of sensor redundancy techniques use hardware and/or analytical programs. Analytical redundancy uses a reference model of the engine to provide redundant estimates of a measured engine variable using a data base from high reliability sensors. Hardware redundancy uses multiple sensors to measure the

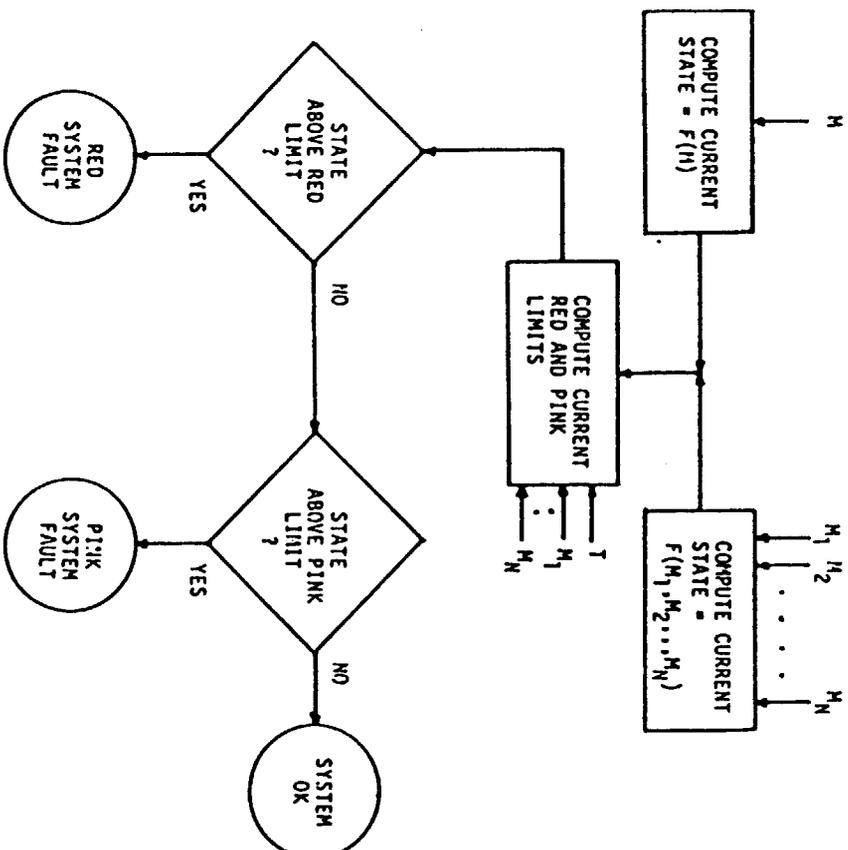


Figure 18. Generalized Detection Algorithm Flow Diagram

## VI, C, Designing for System Reliability (cont.)

same variable. Voting strategies can then be used to detect and isolate sensor failures so that a faulty sensor can be eliminated from the system without initiating a false command. A disadvantage of hardware redundancy is that it is insensitive to the type of failure, hard or soft, since any discrepancy between two redundant sensors always signals a failure. Hard failures are classified as failures that are out-of-range or large-in-range failures, soft failures are small in-range or small bias shifts. Other drawbacks to hardware redundancy are reflected in added component complexity, cost, and weight. Analytical redundancy is able to distinguish hard and soft sensor failures. Range or rate checks are used to provide successful detection of hard sensor failures and statistical decision theory and optimal filtering are used to detect soft failures. A functional block diagram of a sensor fault algorithm is shown in Figure 19. A sample of a detection algorithm is given in Figure 20. State estimate techniques are commonly used for analytic redundancy; however, limitations are imposed by the necessity of an accurate linear model and the assumption that the disturbances on the system are well modeled or else have an insignificant effect on plant or parameter variations. A new approach that eliminates the need for state-space computations uses the estimation error space of each observer in a bank of observers to detect and isolate sensor failures.

A very important benefit of a multiple engine system configuration is the improvement in the redundancy capabilities of the health management system. For example, sensor measurement data from one engine can be used as redundant information for sensor failure detection on another engine. Since proper control response requires comparison of key operating variables from each engine, the interconnection of systems is already necessary, and the benefit to health management is obtained at little extra cost.

In an earlier task a Failure Modes and Effects Analysis (FMEA) was completed for the OTV engine. This analysis was updated and is included as Appendix C as it is still relevant to the baseline engine design. The analysis covers the critical components. Other failures are possible. The list of failure modes in Table IX was generated using the FMEA.

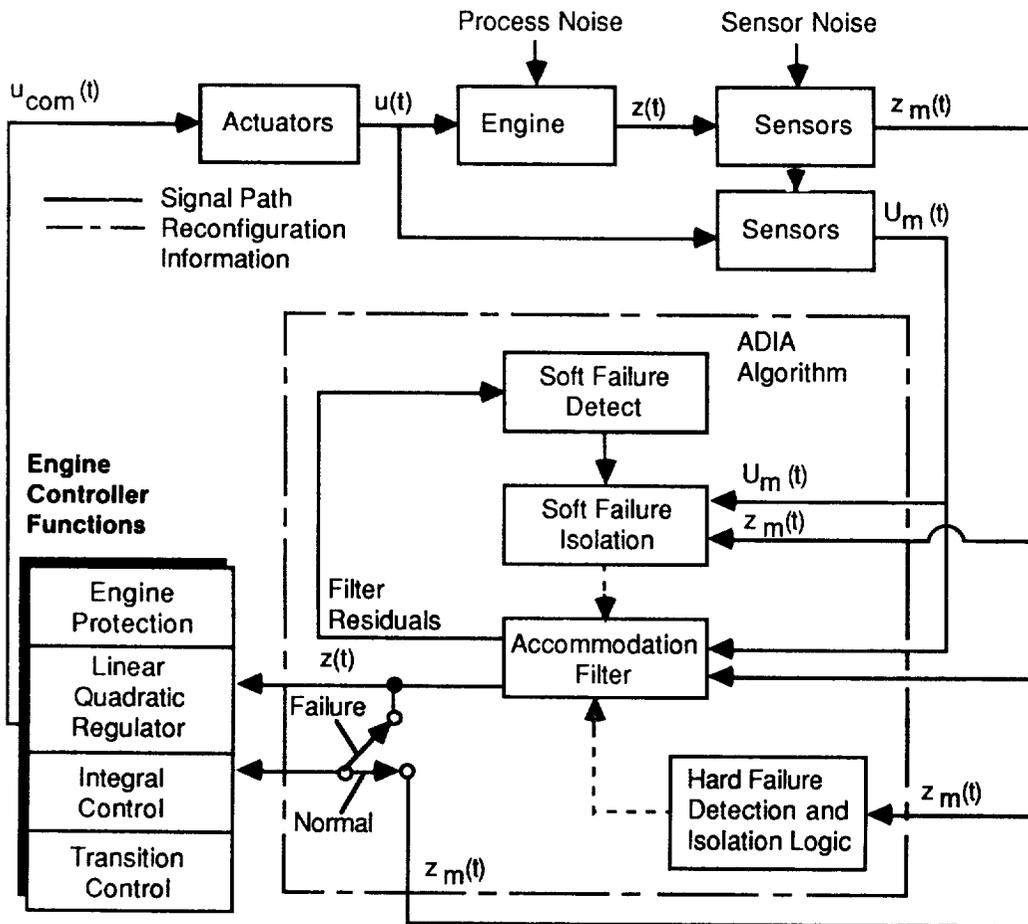


Figure 19. Testbed System with Advanced Diagnosis Algorithm (ADIA)

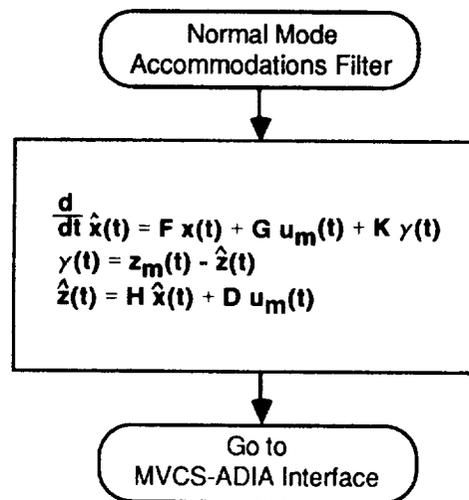


Figure 20. Normal Mode Accommodation Filter Logic

## VI, Health Management (cont.)

### D. MODULAR HEALTH MANAGEMENT SYSTEM DESIGN

The health management algorithms have been classified under three levels of implementation: engine, component and sensor. The general structure is outlined in Table X according to the task performed, algorithm used and available software to perform the task. The functional descriptions of these algorithms at the engine sensor levels is detailed in Tables XI and XII, respectively. Some of the health management task objectives may be limited by the sensor and computational capabilities of the system.

The component failure detection algorithms compare actual to expected component performance. Expected performance is derived from analytical component models, developmental hardware testing, and prior component experience. System measurements are used to determine actual performance. A disparity between actual versus predicted performance signals the health management system to take appropriate corrective actions. An example of fault detection/verification logic for a gas generator-driven TPA is shown in Figure 21.

A block diagram of the basic flow paths of the health management module is shown in Figure 22. The system is subdivided into a set of real-time and off-time functions. Real-time functions which occur during engine operation include: sensor measurement validation, sensor and component failure detection, isolation and accommodation, data acquisition and alarm signaling. Off-time functions are performed during pre- and -post-flight regimes for assessment of engine readiness and maintenance needs.

### E. HEALTH MANAGEMENT DATA STORAGE

A number of possibilities exist for data storage of health management data. Available memory technologies and their associated attributes are summarized in Table XIII.

The health management data recorder requirements differ somewhat from those for the engine controller. More memory storage is necessary; however, the processing requirements for health management data recording are less than

**Table X  
Health Management Algorithm**

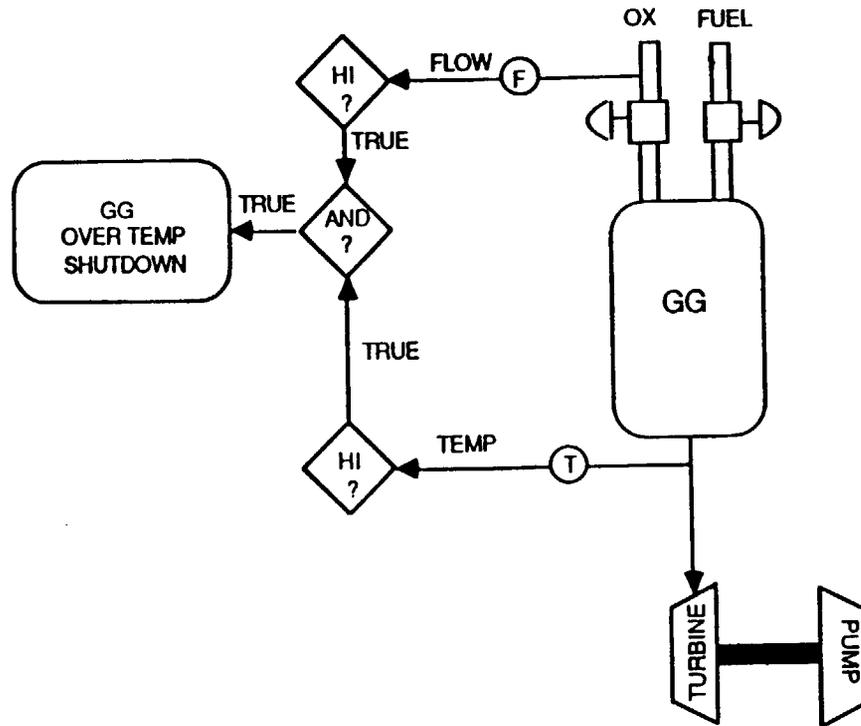
| <u>Level</u> | <u>Task</u>                              | <u>Algorithm</u>                            | <u>Software Type</u>         | <u>Software Code</u> |
|--------------|--|---|------------------------------|----------------------|
| Sensor       | Data Acquisition                         | Protocol                                    |                              | Assembly             |
|              | Signal Conditioning<br>(Preconditioning) | Filter                                      | Band Pass Filter<br>(Simple) | Jovial/Ada           |
| (Component)  | Failure Detection                        | Redundancy Management                       |                              | Jovial/Ada           |
|              | (Component)                              | Level Checking                              | (Simple)                     | Jovial/Ada           |
| (Component)  | (Data Construction)                      | Signature Analysis                          | Time Series, FFT             | Jovial/Ada           |
|              | Sensor Evaluation                        | Kalman Filter                               | Kalman Filter                | Jovial/Ada           |
| Component    | Model Construction                       | Diagnostic Testing/Calibrating/Requalifying |                              | Lisp/Expert          |
|              | State Estimation                         | Status Diagnostic                           |                              | Lisp/Expert          |
|              | Parameter Ident.                         | Kalman Filter                               | Kalman Filter                | Jovial/Ada           |
|              | Trend Analysis                           | Recursive Least Squares                     | Optimization                 | Jovial/Ada           |
|              | System Ident.                            | Regression Modeling                         | Time Series                  | Jovial/Ada           |
|              | Failure Detection                        | Verify Component                            | FFT                          | Jovial/Ada           |
| Engine       | Report/Document                          | Failure Diagnostic                          |                              | Lisp/Expert          |
|              | Engine Reconfig.                         | Condition Prognostic                        |                              | Lisp/Expert          |
|              | Mission Evaluation                       | Data Base Management                        |                              | Lisp/Expert          |
|              | Maintenance Schedule                     | Failure Mode Management                     |                              | Lisp/Expert          |
|              | Engine Command                           | Mission Profile Analysis                    |                              | Jovial/Ada           |
|              |  | Logistic Management                         |                              | Lisp/Expert          |
|              |  | Command Schedule                            |                              | Jovial/Ada           |

**Table XI**  
**Health Management Software at Engine Level**

| <u>Task</u>              | <u>Algorithm</u>         | <u>Function Description</u>  |
|--------------------------|--------------------------|--|
| Model Construction       | Status Diagnostic        | Failure Mode Compensation  |
| State Estimation         | Kalman Filtering         | Estimate State   |
| Parameter Identification | Recursive Least Squares  | Identify System Parameter  |
| Parameter Trend Analysis | Regression Modeling      | Predict Engine Condition   |
| Component Evaluation     | System Identification    | Verify Comp. Character   |
|                          | Level Checking           | Level, Rate Threshold  |
|                          | Signature Analysis       | Pattern Recognition of Component   |
| Failure Mode Detection   | Failure Diagnostic       | Identify Failure Mode  |
|                          | Condition Prognostic     | Identify Health Condition  |
| Report/Document          | Data Base Management     | Manage Health Monitoring Data  |
| Engine Reconfig.         | Failure Mode Management  | Reconstruct Engine Geometry  |
| Mission Evaluation       | Mission Profile Analysis | Evaluate Mission Profile   |
| Maintenance Scheduling   | Logistic Management      | Post Flight Maintenance Schedule   |
| Engine Command           | Command Schedule         | Engine Reconfig. Compensation  |
| Signal Routing           | Protocol                 | Supply User Functions:<br>1) Controller<br>2) Vehicle Operator<br>3) Telemetry<br>4) Health Monitor Recorder |

**Table XII**  
**Health Management Software at Sensor Level**

| <u>Task</u>            | <u>Algorithm</u>  | <u>Function Description</u>   |
|------------------------|---|---|
| Data Acquisition       | Protocol  | Multiplexing<br>Sensor Status<br>Registration<br>Sampling Rate<br>Data Storage                                  |
| Signal Conditioning    | Filter<br>Compensation  | Noise Separation<br>Gain Tuning   |
| Failure Detection      | Redundancy Management<br>Level Checking<br>Signature Analysis | Voting, Averaging,<br>Mid-Value<br>Level, Rate Threshold<br>Pattern Recognition, Self<br>Inspection             |
| Data Construction      | Kalman Filtering  | State Estimation<br>Estimate State  |
| Signal Preconditioning | Filter  | Disturbance Isolation   |
| Recovery Evaluation    | Diagnostic Testing<br>Calibration<br>Requalification          | Testing Sensor<br>Calibrating Sensor<br>Requalifying Sensor   |
| Signal Routing         | Protocol  | Supply User Functions:<br>1) Controller<br>2) Vehicle Operator<br>3) Telemetry<br>4) Health Monitor<br>Recorder |



Note: Example Only; Not an OTV Engine System

LOGIC TABLE

| GG EXIT TEMP | FLOW | CONDITION                        |
|--------------|------|----------------------------------|
| OK           | OK   | GG OK                            |
| HI           | OK   | GG TEMP FAILURE OR OTHER PROBLEM |
| OK           | HI   | FLOW FAILURE OR OTHER PROBLEM    |
| HI           | HI   | GG OVERTEMP                      |

Figure 21. Example of a System Fault Detection/Verification Logic for a Gas Generator Turbine Drive System

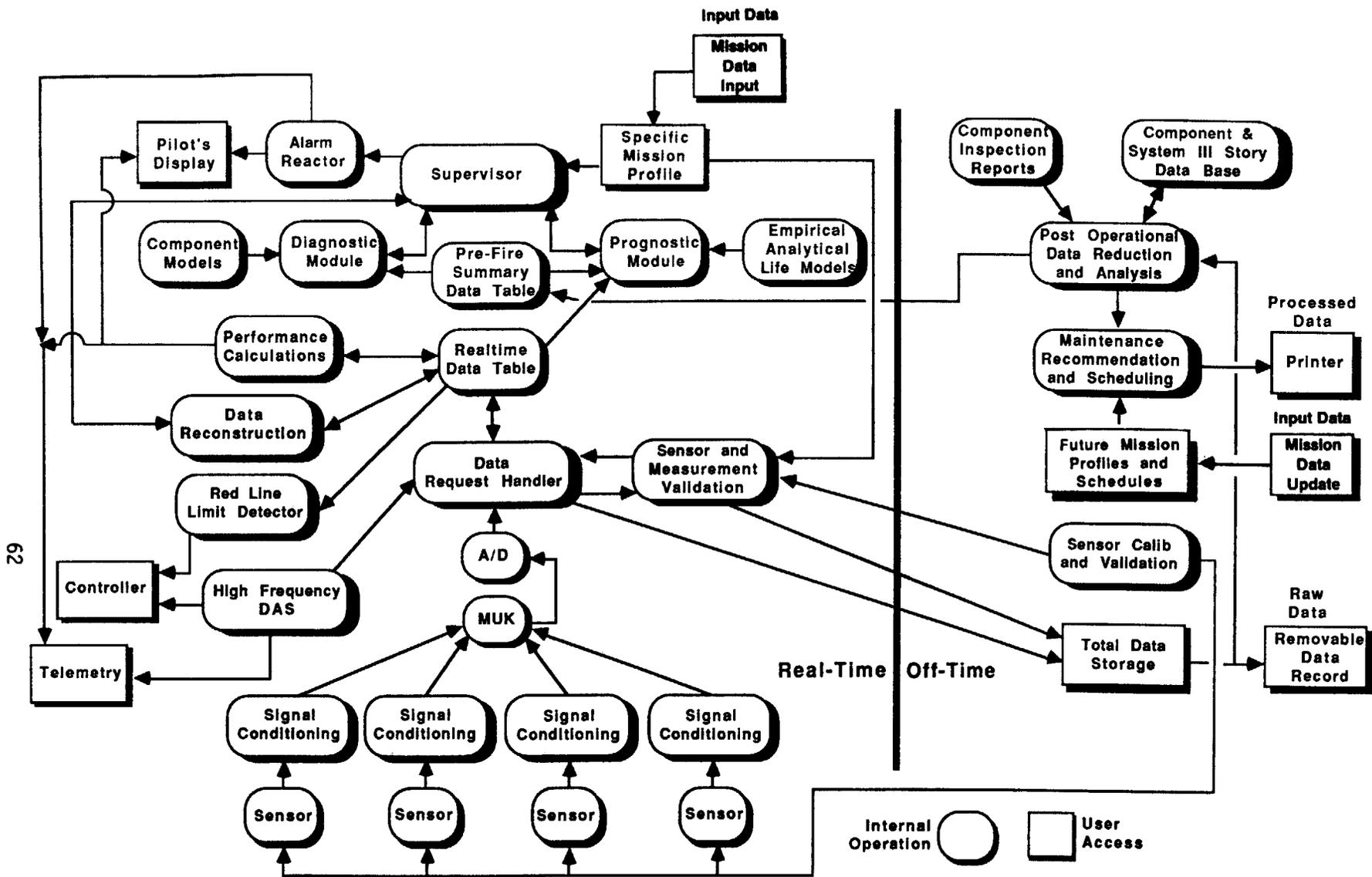


Figure 22. Condition Monitoring Block Diagram

TABLE XIII

AVAILABLE DATA STORAGE TECHNOLOGIES

1. SOLID STATE MEMORY

- Fastest approach possible
- Requires more volume for storage than other technologies
- Very rugged and easy to interface

2. MAGNETIC MEMORY DEVICES

- Magnetic Tape Drives - Extensive use for aircraft and unmanned satellites
- Bubble Memory Devices - Rugged but low density and power hungry
- Ruggedized Hard Discs - High storage capacity and speed

3. OPTICAL DISKS - DERIVED FROM CONVENTIONAL COMPACT DISK TECHNOLOGY

- Write once read only memory (WORM)
- Gaining acceptance for use in military aircraft
- Very high data capacity (1/2 gigabyte per 5" disk)

TRADES REQUIRED TO MAKE DECISION

- Total data traces to be recorded
- Sample rates of the data traces
- Susceptibility to radiation environment

## VI, E, Health Management Data Storage (cont.)

that for the real-time control computations of the controller. Single-string electronics are sufficient for non-flight critical functions and there are essentially no output electronics required for the data recorder.

Selected data streams will be processed and used for pilot display. At a minimum there will be a real time presentation of the following information for each engine:

1. Position in the start or shutdown sequence
2. Engine thrust
3. Engine mixture ratio
4. Turbine inlet temperatures (O<sub>2</sub> and H<sub>2</sub>)
5. Gimbal position
6. Engine compartment temperature
7. Extendible exit cone position
8. Fuel and Ox start valve position
9. Turbine speed (O<sub>2</sub> and H<sub>2</sub>)

Operation of a piloted space vehicle will be closely monitored by the spaceport and earth-based stations. Engine condition will be part of the routine telemetry broadcast from the OTV. The telemetry will include normal operational data but will be expanded to add data on red line or emergency conditions encountered during a mission. This is real-time data and may differ from the recorded data and pilot display information.

## F. APPLICATIONS FOR ARTIFICIAL INTELLIGENCE AND EXPERT SYSTEMS

A comprehensive health management system will produce an enormous amount of data. At any given time most of it is redundant or irrelevant. Discrimination between pertinent and useless data requires informed judgement. Traditionally this is where a human takes over from the machine, but any person or even a whole crew will be unable to handle the mass of data for real time decision making. Data selection will have to be a function of the computer with only limited crew-directed additions. The computer should be capable of bringing a potential problem to the crew's attention, suggesting solutions, evaluating pilot input commands, and presenting alternatives should the pilot input be predicted to worsen

## VI, F, Applications for Artificial Intelligence and Expert Systems (cont.)

the problem. When time does not permit dialog with a human the computer must act to save the mission and/or the vehicle while informing the crew of the problem and the corrective action taken. These capabilities are at the current state-of-the-art, and will have to be developed for future manned space travel. The use of expert systems software will be needed for these demanding applications.

### G. HEALTH MANAGEMENT SENSORS

The group of sensors required for basic engine control was described previously and is shown in Table I. Additional sensors are used to supplement the control sensors for the purpose of health management. Specifically, they are used for failure detection, isolation and accommodation, performance monitoring, and data acquisition of non-critical flight parameters used for post-flight maintenance analysis. A combined list of control and health management sensors is presented in Table XIV. A more detailed listing is given in Appendix B.

A review of advanced technology was made for bearing monitoring, pressure measurement, shaft monitoring, and hot spot detection. Bearings are critical life-limiting components in rocket engine turbopumps, thus bearing monitoring is an important facet of the health management system. The function of shaft monitors is to measure axial and radial shaft deflection and shaft speed all of which may be used for TPA diagnostics. Preliminary requirements for a displacement sensor are given in Table XV.

Currently silicon microchip displacement sensors are one of the more effective means of measuring rolling contact bearing outer race deflection. This measurement gives a direct indication of bearing health. Race deflection is sinusoidal in nature due to the balls passing by the sensing element as they travel around the race, refer to Figure 23. The signatures created by the balls are analyzed to determine bearing defects. By comparing the signatures of a good bearing with a bearing containing a defective ball, Figure 24, it is evident that this method provides a straightforward means of fault detection.

The objectives of investigating advanced sensor technologies for shaft monitoring are: to provide significant size and/or weight reduction, reliable operation at cryogenic temperatures, and improved packaging of signal conditioning

TABLE XIV

CONTROL AND HEALTH MANAGEMENT SENSORS

| <u>Component</u> | <u>Required Measurements</u>   | <u>Function</u>   |
|------------------|--|---|
| Pump             | Flow Rate<br>Shaft Speed<br>Inlet Pressure<br>Discharge Pressure   | Head, Flow, Speed<br>Relationships and<br>Cavitation Effects    |
| Turbine          | Shaft Speed<br>Inlet Pressure and Temperature<br>Outlet Pressure and Temperature<br>Shaft Vibration<br>Turbine Blade Temperature<br>Turbine Blade Clearance                                    | Turbine Performance<br>Verification<br><br>Turbine Blade Health |
| Bearing          | Bearing Supply Pressure<br>Bearing Temperatures<br>Hydrostatic Channel Flow Rates<br>Radial Shaft Motion Transducers<br>Axial Shaft Motion Transducers<br>Shaft Speed                          | Bearing Health,<br>Efficiency                                   |
| Thrust Chamber   | Coolant Jacket Inlet and<br>Outlet Pressures<br><br>Coolant Jacket Inlet and<br>Outlet Temperatures<br>Chamber Pressure<br>External Coolant Jacket Wall<br>Temperatures for Hot-Spot Detection | Thrust Chamber<br>Health and<br>Performance                     |

TABLE XIV (cont.)

| <u>Component</u> | <u>Required Measurements</u>  | <u>Function</u>                       |
|------------------|---|---------------------------------------|
| Control Valves   | Valve Upstream and Downstream Pressures<br>Valve Position<br>Acoustic Internal Leak Detectors<br>"Sniffer-Type" External Leak Detectors | Valve Monitoring                      |
| Injector         | Injector Inlet Pressure<br>Chamber Pressure<br>Injector Inlet Temperature<br>Face Temperature   | Health and Performance                |
| Igniter          | Ignition Pulse Detector<br>Cavity Temperature   | Operation Verification,<br>Cooling    |
| Regen Liner      | Temperature Distribution  | Health and Trend (Life)<br>Prediction |

TABLE XV

DISPLACEMENT SENSOR REQUIREMENTS

|                           |   |
|---------------------------|---|
| Type                      | TBD   |
| Range                     | 0.001 to 0.005 inch radial<br>0.001 to 0.009 inch axial |
| Resolution                | 0.00001 inch  |
| Error Band                | ± 1.0% of range   |
| Repeatability             | ± 0.25% of range  |
| Thermal Sensitivity Shift | ± 0.005% of range/°C                                    |
| Temperature Range         | -253°C to +100°C  |
| Frequency Response        | 50 KHZ  |
| Pressure Range            | 0 to 10,000 psia  |
| Excitation                | 10 volts DC   |
| Signal                    | 0 to 5 volts DC   |
| Size                      | TBD   |
| Material Compatibility    |   |
| Liquid & Gaseous          | Oxygen  |
| Liquid & Gaseous          | Hydrogen  |
| Liquid & Gaseous          | Nitrogen  |

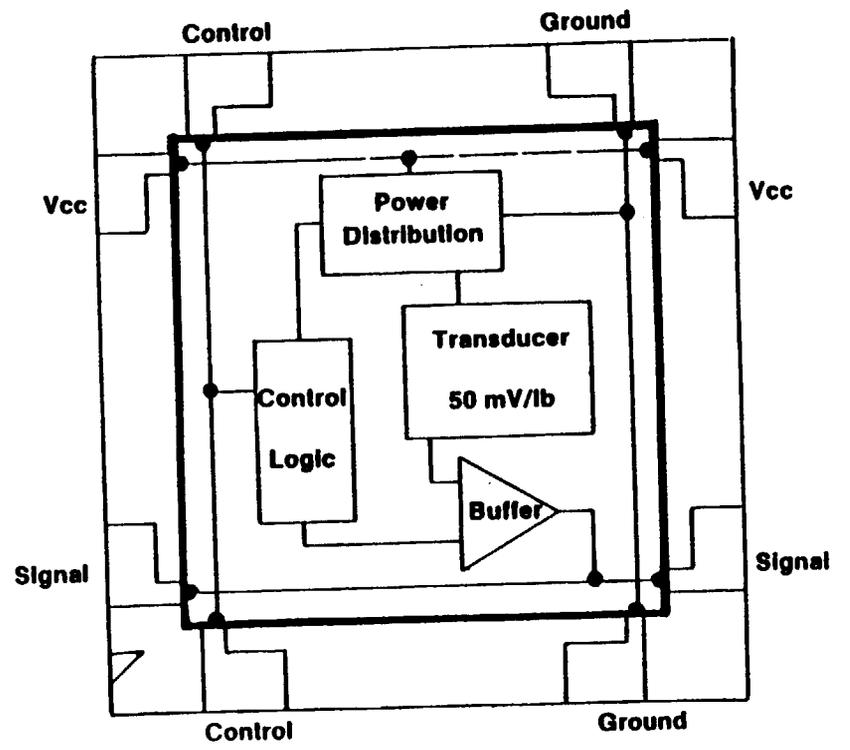
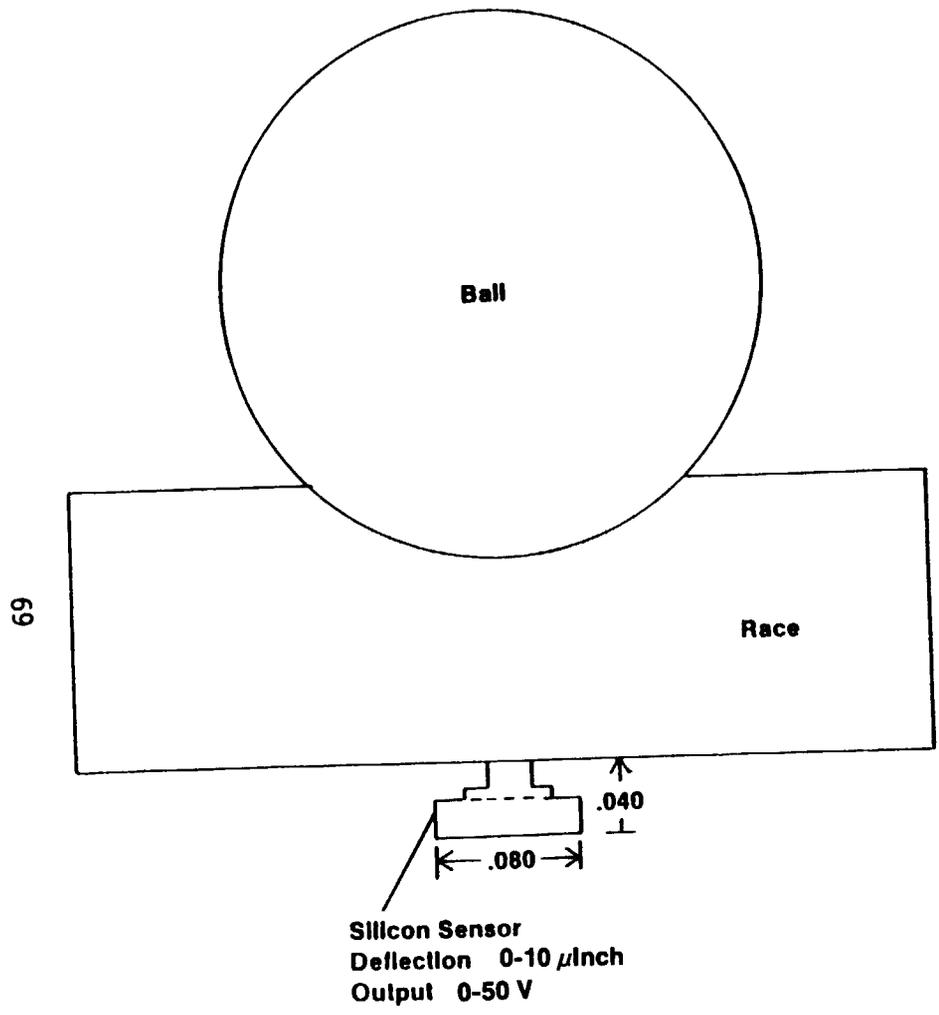
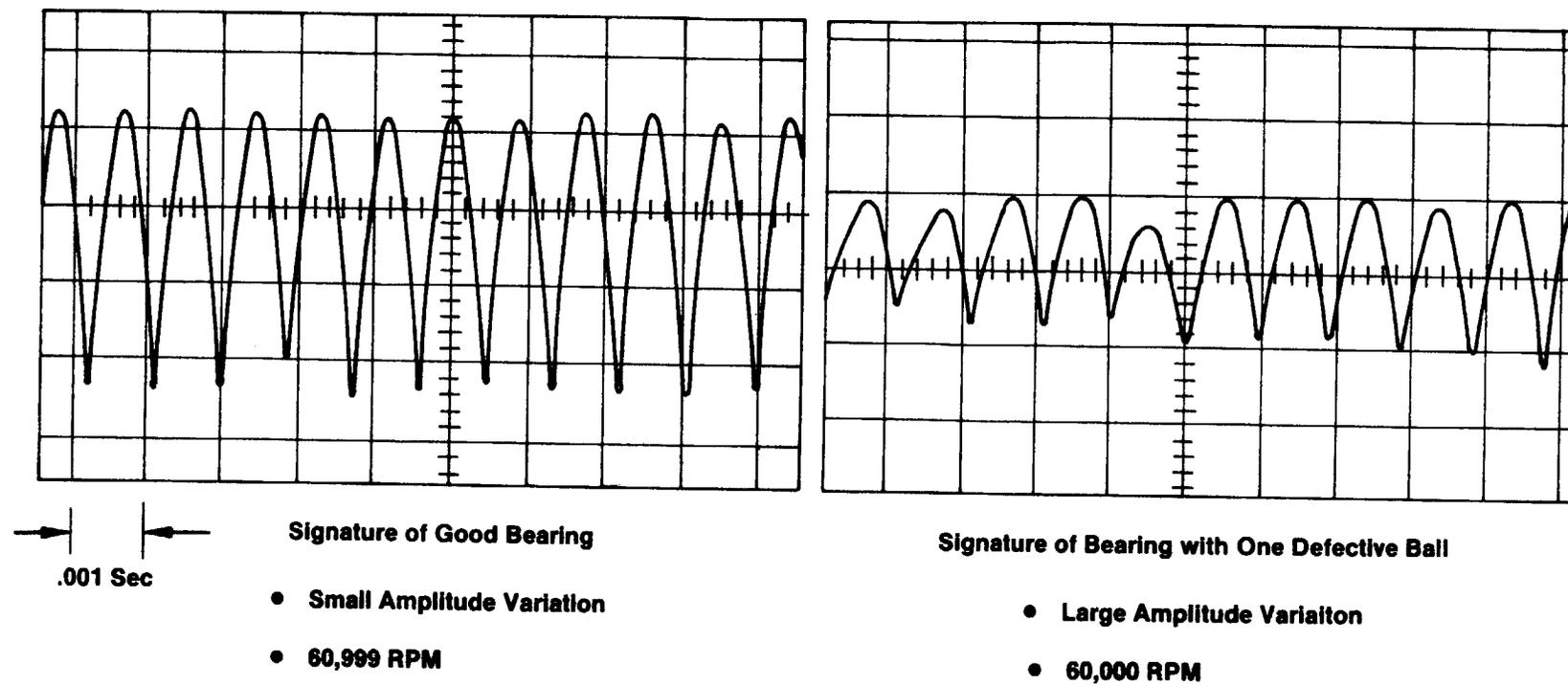


Figure 23. Example of a Bearing Health Monitoring Silicon Sensor



**Figure 24. Reproduction of Actual Oscilloscope Traces of Turbopump Bearing Signatures**

## VI, G, Helath Management Systems (cont.)

logic. The current approach involves the use of a multifunction sensor probe for axial and radial displacement and speed. Methods which seem most likely to meet the accuracy and resolution requirements of the application are capacitive, optical (fiberoptic) and inductive. The capacitive transduction method appears to be the most stable under varying environmental conditions as capacitance is a function of the physical dimensions and dielectric constant of the media. A further advantage is that the capacitive sensor tip is the most adaptable to multiple function sensing. A conceptual design based on a three function sensor using capacitive transduction has been prepared and is in development. Figure 25 is the sensor envelope which is designed to fit the oxidizer turbopump. Figure 26 shows the active sensor tip configuration. The sensor tip has three sets of sensing electrodes, one set for radial displacement and two sets for axial displacement. One set of axial displacement electrodes is used for speed pickup when a step is machined into the running surface. The sensing tip is ceramic with metallized electrodes. Signal processing electronics are provided in a semiconductor chip at the sensor tip. Local signal processing is essential to minimize the effects of leakage capacitance.

An electronic hot spot detector for measuring combustion chamber cooling channel temperature is shown in Figure 27. The detector is composed of 120 equally spaced thermocouples (refer to Figure 28 for the sensor configuration). This sensor can be used for the direct detection of local or overall chamber overheating.

### H. SENSOR SUPPLIER SURVEYS

ATC has conducted two surveys of health monitor suppliers. The first was a comprehensive survey completed in 1983. The survey data is presented in Figure 29. Approximately 40% of the companies contacted responded. Most of these had product lines closely related to commercial operations and applications. ATC did select 19 suppliers as producing or capable of producing sensors of use in the OTV engine.

The second survey was conducted in February 1987 to assess the available technology for sensing high speed turbopump shaft motion. An initial literature search yielded 122 items related to displacement sensor application. Detailed literature was ordered and evaluated. In addition, the profiles of 151 companies listed in



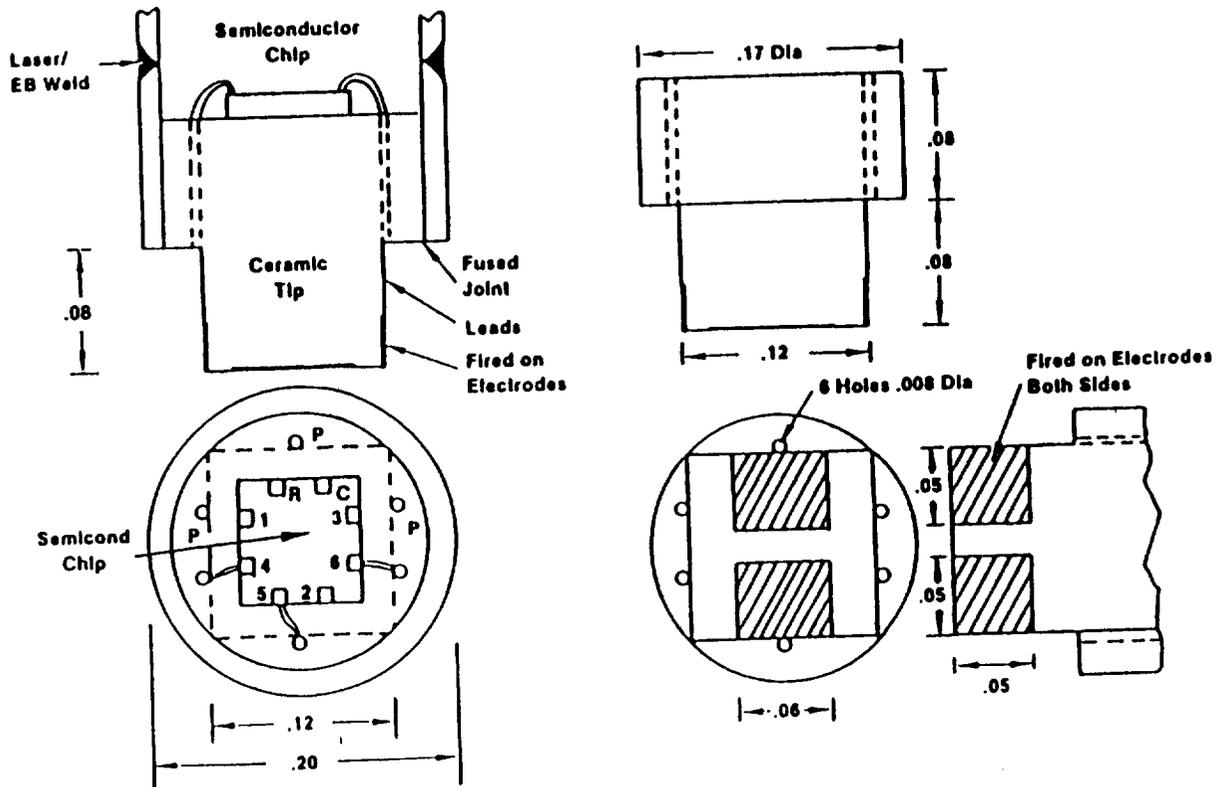
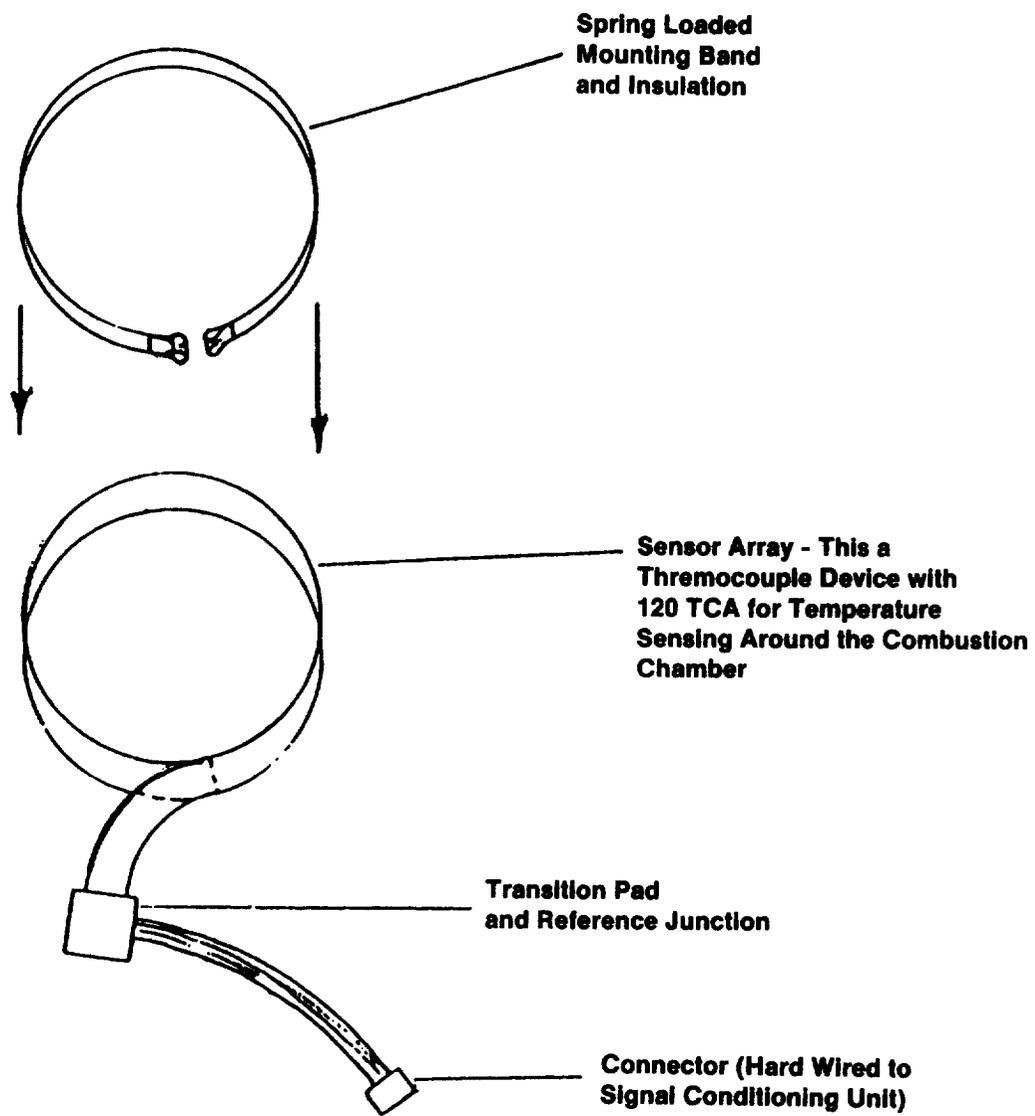


Figure 26. Multifunction Turbopump Shaft Monitor Active Tip Configuration



**Figure 27. Hot Spot Sensor Assembly**

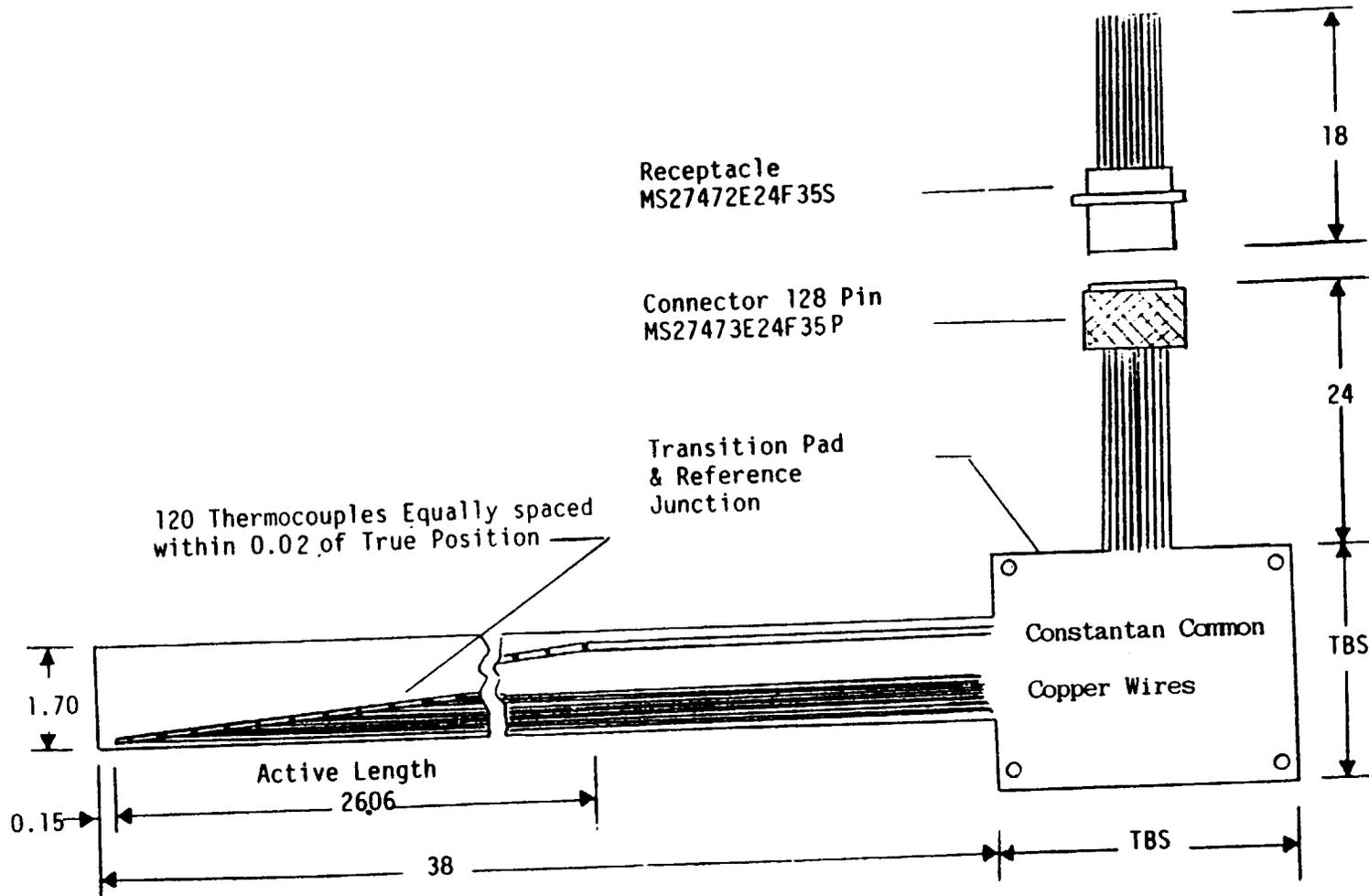


Figure 28. Hot Spot Sensor Configuration

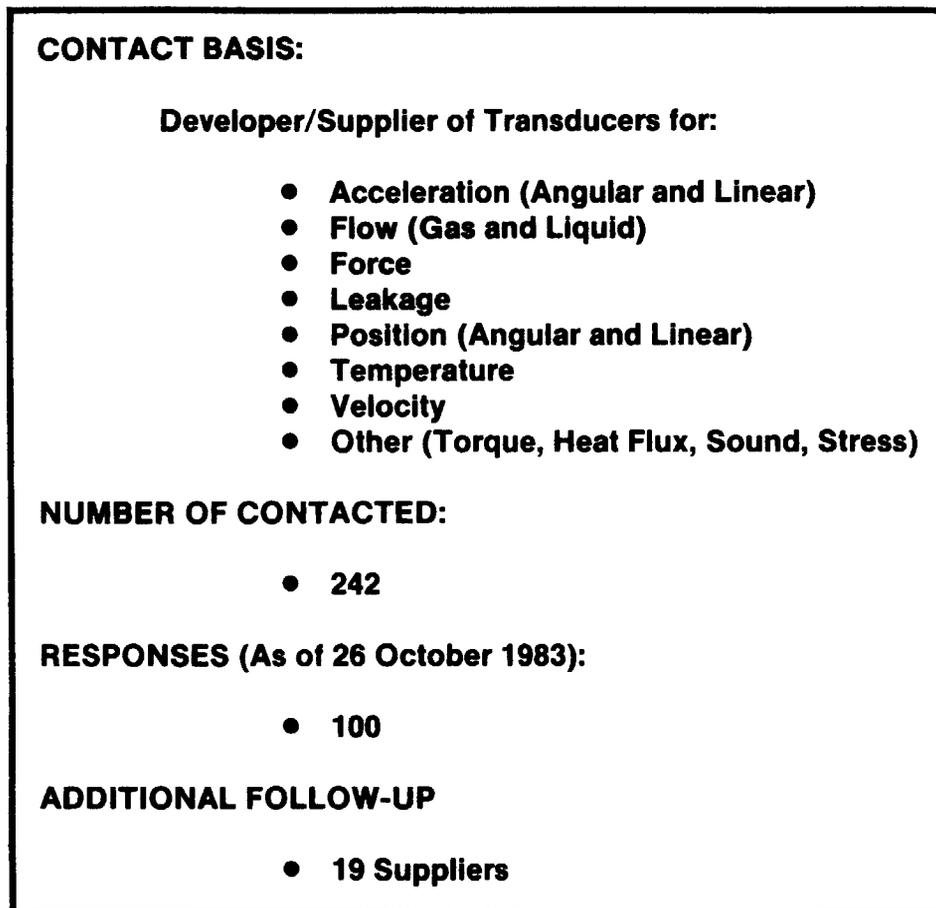


Figure 29. OTV Engine Health Monitoring - Sensor Supplier Survey

VI, H, Sensor Supplier Surveys (cont.)

the Sensor and Transducer Directory were reviewed. They were all listed as having displacement or proximity sensors in their product line. Inquiries were sent to 50 of these companies asking for information on the following types of sensors:

- Displacement, linear
- Displacement, mechanical
- Displacement, optical
- Displacement, ultrasonic
- Proximity, magnetic
- Proximity, ferrous
- Proximity, non-ferrous
- Proximity, ultrasonic
- Proximity, capacitive

On examination of the literature received from the responding companies, the methods judged most likely to meet the OTV engine turbopump sensor requirements are capacitive, optical (fiberoptics), and inductive. the requirements for a displacement sensor are given in Table XV.

The results of the supplier survey are included as Appendix A.

I. REVIEW OF VEHICLE PRIME STUDIES

The vehicle primes have produced many volumes documenting various concepts and analytical results as a prelude to actual OTV development. Very little of this mass of information is directly applicable to problems of engine control and health monitoring. What is relevant to the propulsion designer are 1) the type of missions in the mission model, 2) total vehicle propellant loads, 3) the type of aerobraking proposed, 4) maintenance concept and timelines (to include delivery, assembly, checkout, disassembly, and preparation for return to earth), and 5) life cycle cost implications. To some extent these mission requirements have been distilled and quantified as the propulsion requirements presented in Table II. The most notable omission is a listing of which engine components should be replaceable in space, and any type of criteria for engine time change. The ATC 3.0K thrust engine was designed for replacement at the engine level with no on-the-vehicle engine maintenance. Thus any anomalous health monitor system diagnosis would

## VI, I, Review of Vehicle Prime Studies (cont.)

trigger an engine change. This is a very desirable situation for decision making but could be costly if engine changeout required significantly more time than a component changeout.

The present ATC approach recognizes the vehicle Prime contractor's quantification of the high cost of in-space maintenance and the desirability of component versus engine changeout. The 7.5K thrust engine will be designed for some component changeout. These components are listed in Table XVI. This is a fairly long list and may be incomplete at this stage in the engine design. The O<sub>2</sub> and H<sub>2</sub> turbopumps will be designed for changeout in space although it may be more economical to perform the changeout with the engine removed to a specialized maintenance facility at the spaceport. The concern is to preserve the integrity of the lines to and from the pumps. When work will be done with little clearance and with all the restrictions of space suit operation it could prove cheaper to replace an engine than struggle with a difficult component changeout. Timelines must be developed for each maintenance operation to aid in the decision.

The General Dynamics vehicle study did address delivery, assembly, and OTV servicing operations. Table XVII is a listing of the major operations from arrival at the Space Station to payload integration or storage of the OTV. Those operations where the engine health monitor system will be used are indicated by an asterisk. The typical operational scenario for an OTV mission is divided into time sequenced functions as indicated in Table XVIII. The engine health monitor system is used in those functions indicated, as before, by an asterisk. A more detailed outline of the space based maintenance functions is given in Table XIX. The health monitor system will be used for both troubleshooting and verification that the maintenance has corrected the problem.

Of the five items listed above as relevant to the propulsion designer, the most demanding is that of man-rating. The health monitor and control system must assure safe piloted operation of the OTV in so far as this is a function of the propulsion system. If this is built into the design most of the other requirements will be satisfied by straightforward engineering solutions. Low life cycle cost is primarily a function of engine life and overall reliability. ATC is designing for low stress levels on all parts, for wide strength margins in all high temperature

TABLE XVI

OTV ENGINE SPACE MAINTAINABLE COMPONENTS  
7.5 LBF THRUST ENGINE

COMMENT

COMPONENT

HYDROGEN MAIN SHUTOFF VALVE  
OXYGEN MAIN SHUTOFF VALVE  
HYDROGEN BOOST PUMP (LOW PRESS)  
OXYGEN BOOST PUMP (LOW PRESS)  
HYDROGEN AUTOGENOUS PRESSURIZATION VALVE  
OXYGEN AUTOGENOUS PRESSURIZATION VALVE  
HYDROGEN REGENERATOR BYPASS VALVE  
OXYGEN REGENERATOR BYPASS VALVE  
GIMBAL MOTORS  
GIMBAL ACTUATORS  
EXTENDIBLE NOZZLE  
EXTENDIBLE NOZZLE DEPLOYMENT MOTORS  
EXTENDIBLE NOZZLE DEP. MECHANISM  
FUEL FLOWMETERS  
OXYGEN FLOWMETERS  
CONTROLLER  
SENSOR SIGNAL CONDITIONING UNITS  
  
MISCELLANEOUS HARDWARE, BRACKETS, WIRES,  
EXTERNAL SENSOR ELEMENTS

REQUIRES ACCESS NEAR ENGINE/VEHICLE INTERFACE  
DESIGN DEPENDENT; MAY BE A BOLT-ON TO A MANIFOLD  
DESIGN DEPENDENT; MAY BE A BOLT-ON TO A MANIFOLD  
REQUIRES ACCESS NEAR ENGINE/VEHICLE INTERFACE  
REQUIRES ACCESS NEAR ENGINE/VEHICLE INTERFACE  
READY ACCESS  
REQUIRES ACCESS NEAR ENGINE/VEHICLE INTERFACE  
REQUIRES ACCESS NEAR ENGINE/VEHICLE INTERFACE  
DESIGN DEPENDENT; MAY BE BOLT-ON TO LOW PRESS BOOST PUMP  
DESIGN DEPENDENT; MAY BE BOLT-ON TO LOW PRESS BOOST PUMP  
ONE OR TWO REMOVABLE BOXES WITH CANNON PLUGS  
DESIGNED TO ALLOW SENSORS TO REMAIN IN PLACE, ONLY  
ELECTRONICS CHANGED (REQUIRES SYSTEM RECALIBRATION)  
DEPENDENT ON ACCESS

TABLE XVII

INITIAL OTV DELIVERY AND ASSEMBLY OPERATIONAL FUNCTIONS

Dock Shuttle to Station

- Offload and position core section
- Offload aerobrake, struts, aerobrake door, and RCS thrusters
- Deploy aerobrake
- Attach RCS thrusters, aerobrake, and aerobrake door
- Inspect OTV assembly

Checkout OTV System

- Bring all systems online\*
- Perform OTV system operational testing\*
- When fault or damage detected\*
  - Perform damage assessment
  - Initiate fault isolation routine
  - Perform OTV unscheduled maintenance
  - Perform OTV system operational testing
- Proceed with payload integration or deactivate and stow OTV

\*Operations where the OTV engine health monitor system is used

TABLE XVIII

OTV SERVICING OPERATIONAL FUNCTIONS

BERTH OTV

- RENDEZVOUS OTV WITH STATION
- CAPTURE OTV AT STATION
- BERTH OTV AT STATION

TRANSFER PROPELLANT

- VERIFY INTERFACE INTEGRITY
- PERFORM PROPELLANT LEAK CHECK
- TRANSFER RESIDUAL PROPELLANT FROM OTV STATION

INSPECT OTV

- PERFORM VISUAL INSPECTION
- DETERMINE OTV FAULT STATUS\*
- WHEN FAULT OR DAMAGE DETECTED\*
  - PERFORM DAMAGE ASSESSMENT (TV/EVA)
  - INITIATE ELECTRICAL TEST ROUTINE TO VERIFY FAULT
  - INITIATE FAULT ISOLATION ROUTINE
- FORMULATE INTEGRATED MAINTENANCE PLAN\*

PERFORM OTV MAINTENANCE

- PERFORM SCHEDULED/UNSCHEDULED MAINTENANCE TASKS\*
- MISSION RECONFIGURE
- PERFORM SYSTEM OPERATIONAL TESTING\*
- DEACTIVATE & STOW OTV (IF NOT REQUIRED FOR MISSION AT THIS TIME)

MATE OTV & PAYLOAD

- TRANSFER PAYLOAD TO OTV
- MATE PAYLOAD TO OTV
- VERIFY OTV/PAYLOAD INTERFACE
- PERFORM OTV/PAYLOAD INTEGRATION TEST

LAUNCH OTV/PAYLOAD

- PERFORM PRELAUNCH OPERATIONS\*
- TRANSFER PROPELLANT FROM STATION TO OTV
- LAUNCH OTV/PAYLOAD\*

\*OPERATIONS WHERE THE OTV ENGINE HEALTH MONITOR SYSTEM IS USED

TABLE XIX

SPACE-BASED OTV MAINTENANCE FUNCTIONS

Perform scheduled maintenance

- Transfer propellant to and from OTV
- Perform visual inspection
- Determine OTV fault status
- Replace ACS modules (after each mission)
- Replace engine module\* (after TBD mission time)
- Perform system operational testing
- Service fuel cell (after TBD mission time)

Perform unscheduled maintenance

- Perform damage assessment (beyond scheduled inspection)
- Verify electrical failure
- Isolate fault to replaceable unit
- Perform damage repair
- Perform required remove and replace due to failure

-----  
\*The vehicle can be developed with replaceable propulsion modules or for individual engine replacement. In-space handling requirements may dictate one or the other design solution. Table XX lists the steps in removing an engine in Aerojet's design concept.

TABLE XX

OTV ENGINE REMOVAL OPERATIONS

- Engine Centered, Nozzle Extended
- Propellant Isolation Valves Closed
- Electrical Power Removed from the Engine
- Manual Operations:
  - Electrical Harness Disconnected, Cannon Plugs Capped, Harness Stowed
  - Extendible Nozzle Removed, Screw Assemblies Secured, Regen Cooled Nozzle Edge Protector Installed
  - Main Hydrogen Line Disconnected Below Shutoff Valve, Capped
  - Main Oxygen Line Disconnected Below Shutoff Valve, Capped
  - Hydrogen Tank Autogenous Pressurization Line Disconnected Below Shutoff Valve, Capped
  - Oxygen Tank Autogenous Pressurization Line Disconnected Below Shutoff Valve, Capped
  - Engine Handling Fixture Connected
  - Engine Out Gimbal Actuator Disconnected
  - Control Gimbal Actuators Disconnected
  - Flex Lines Restrained, Upper Engine Covered with Protective Material
  - Prime Mover Connected to Engine Handling Fixture
- Engine Moved out of Engine Compartment

VI, I, Review of Vehicle Prime Studies (cont.)

materials, and the generally conservative design selections that accompany high reliability. The basic control system will be proven on an operational rocket engine 15 years before it is flown on the OTV engine. The health monitor sensors will all have a pedigree that includes a wide range of missions and test conditions. This design approach should give the lowest life cycle cost and the safest operation for a piloted space vehicle.

## VII. DISCUSSION OF RESULTS

### A. ENGINE CONTROL AND MODELING

A variety of control simulations of the dual expander engine design were successfully run, but project changes and resource constraints precluded performing simulations over the entire operating envelope. Such simulations would need to be run if work continues in developing the dual expander cycle engine. Also, the component models used in the simulations can and should be updated as component development proceeds.

The results of the control simulations using the TUTSIM program were acceptable even though more cases would have been desirable. In both throttle-up and throttle-down scenarios the proposed control valve arrangement performed within acceptable limits of mixture ratio and thrust excursion. There was no set-point hunting, and transitions were free from spikes or short term oscillations. The concept of mechanically separate turbopumps for this engine is practical from a controls standpoint from the pumped idle point to some overthrust point above the nominal 7500 lbf of thrust. What could not be proven in the controls simulation is the practicality and controllability of a tank head start. With the very low pressure available (15 psia) for a tank head start actual components, as built versus as designed, can seriously effect operating margins. No one can guarantee a successful tank head start for an engine of this complexity until actual components are built and tested; even then, start-to-start repeatability will be questionable and subject to lack of success from even small changes in system fluid properties or temperatures of components.

Engine shutdown is a much more straightforward operation. The main propellant valves are simply closed and residual propellants allowed to burn to a pressure where combustion is no longer sustained. The relatively large line and component volumes will produce a tailoff impulse lasting 2 or 3 seconds. Both startup and shutdown are slow compared to storable propellant rocket engines where these transient conditions are measured in milliseconds.

## VII, A, Engine Control and Modeling (cont.,

There is a maximum time-rate-of-change for throttle commands that must be compatible with mission requirements. If the most demanding mission is a landing on the moon or Mars then this is calculable once the vehicle weight and number of engines are determined. This should be done early enough in the engine development to establish it as a controls requirement. Otherwise it could become a mission limitation or an expensive problem needing solution. This controls analysis did not attempt to assess the maximum throttle rate with the baseline control system.

The software package used for this simulation, called "TUTSIM," is in widespread use but has several limitations. A major one is incompatibility with frequency domain analysis techniques. It is also primarily adapted for personal computers and the limitations in processing speed and memory of PCs. A main-frame software package such as MATRIX-X includes all linear system analysis functions including interactive classical and modern control design, conversion between model forms, and computation of time and frequency responses. It should be considered for any future extension of this controls modeling work. It was just recently added to ATC's library of programs; too late to be used in this task.

### B. CONTROL ELEMENTS

The basic control elements are valves, flowmeters, pressure and temperature sensors, turbopump speed sensors, and an electronic controller. In the integrated approach sensor data from an additional array of sensors supplements the control sensor data to allow a health management software/firmware system to evaluate engine health and, if necessary, direct a specific controller action. It can also synthesize data for the controller should some control sensor(s) be inoperative. The desired result is a more flexible control system with greatly improved safety and reliability. Work done in this task tends to confirm this as an achievable goal in so far as the mechanical control elements are concerned. Valves can be developed and sensors obtained without any major technology challenges to the OTV engine development. Development work is needed on turbine bypass valves. Sensors need to be miniaturized while reliability is improved. Those are achievable with

VII, B, Control Elements (cont.)

logical extensions of present technology; no breakthroughs are needed. This is not necessarily true of the software/firmware that takes the sensor data and processes it to predict engine health and diagnose engine problems. This will be a significant development task and one that may require artificial intelligence software for best applicability to the task. It is worthy of separate consideration as a development task. The work done here on sensors and system architecture can serve as a good starting point.

C. HEALTH MONITORING

Health management comprises three separate tasks: real time monitoring of engine health; and recording of information for a permanent maintenance record with guidance on component changeout, recalibration, and troubleshooting, and an intelligent selection and processing of data streams. Health management must be done with the speed needed to prevent imminent failure from becoming catastrophic. It is evident that computational requirements are high and programming an extensive process. The basic engine controller can be readily developed from well advanced work on a bipropellant engine controller at ATC. The interacting health management system is not well defined in terms of hardware. It should be an electronic box much like the controller, mounted adjacent to it, have direct connection to the Health Monitor sensors, and have a remotely located recording unit accessible to both vehicle crew and maintenance. The form taken to record its programs is uncertain as is the memory storage. It will be a very sophisticated computer, and should be a candidate for early development work.

## VIII. SUMMARY OF RESULTS

This report documents the first extensive work done in defining an integrated health management and control system for the Orbit Transfer Vehicle (OTV). The work was divided among four subtasks: 1) engine system modeling, 2) control logic evaluation, 3) control element requirements, and 4) health monitoring. The task work led to the following conclusions:

### 1. Engine System Modeling

- a) There is a maximum time-rate-of-change for throttling operation, but additional work needs to be done to define it.
- b) Changes in the details of the engine schematic need to be modeled and incorporated into the system model to assure controllability and stability are maintained as the design evolves.
- c) Additional work needs to be done to tune the systems so that the linear and nonlinear models match.
- d) The most poorly defined engine operation is tank head start. The low pressure budget makes actual equipment variables critical in defining whether it is possible. Actual tests will be needed to prove its feasibility.

### 2. Control Logic Evaluation

- a) The baseline engine control system can control thrust and mixture ratio from pumped idle mode to some thrust above nominal.
- b) Mechanically separate hydrogen and oxygen turbopumps can be used for normal fixed thrust operation and throttling operations.
- c) The control logic evaluation showed that turbine bypass valves could control thrust and mixture ratio as planned over most of the thrust range using flowmeter data. At low thrust conditions, flowmeter data may have to be supplemented with turbine speed readings to assure that mixture ratio does not exceed allowable limits if only one pump should rotate.

VIII, Summary of Results (cont.)

d) Closed loop control configuration is required for the OTV engine. There is no capability for a start without a functioning controller.

e) Control system stability was achieved at all the operating points checked, but additional work needs to be done to assure that the system was stable over the entire operating range. In particular, the range where the oxygen turbine bypass control effects are non-linear needs to be assessed.

f) A preliminary failure-mode and effects analysis (FMEA) was completed for the baseline controls configuration.

g) A reliability assessment was completed in terms of design impact. This report recommends analytical redundancy wherever possible instead of hardware redundancy.

h) Control of this engine is considered verified on a preliminary basis, but additional work should be done concurrent with a test bed program.

i) An integrated approach to health management and control is considered practical with this report serving as a starting point for follow-on work.

3. Control Element Requirements

a) The basic control elements of valves, flowmeters, and sensors have been identified and well enough specified to begin procurement or detail design actions.

b) An ATC developed controller was evaluated for the controller requirement and was found to be acceptable with some modifications. This could be a significant savings in an engine development program.

c) A sensor supplier survey was completed and the results presented. All expected sensor needs can be met with development programs well within the engine development time span.]

VIII, Summary of Results (cont.)

4. Health Monitoring

a) The health management system was defined in sufficient detail that sensors were identified, a system architecture was developed, and a data recording system proposed. This is sufficient definition to be used as the baseline for a detailed design or to assist in defining requirements for a health management system software development program.

b) Prime contractor vehicle studies were reviewed and health management related activities identified. The system will be used for pre-flight, post-flight, and permanent maintenance record purposes based on Prime identified operations.

c) A component in space maintenance list was developed but no timelines were available for evaluating the practicality of component changeout. The alternative of complete engine removal was presented and a list of steps in the changeout included. The basic maintenance philosophy needs to be defined after considering the timelines and extra-vehicular activity (EVA) costs.

IX. REFERENCES

1. Anon,Orbit Transfer Vehicle (OTV) Advanced Expander Cycle Engine Point Design Study. Contract NAS 8-33574 Final Report. Aerojet Liquid Rocket Co. report no. 33574-F for Marshall Space Flight Center, December 1980.
2. Turnaround Operations Analysis Report for OTV, Vol. I, II, III, and IV, Contract NAS 8-30924 Final Report. General Dynamics Space Systems Report No. GD55-SP-87-018, for NASA Marshall Space Flight Center, February 1988.



APPENDIX A  
SENSOR SUPPLIER SURVEY RESULTS

## SURVEY OF AVAILABLE SENSOR TECHNOLOGY

In order to identify and evaluate approaches to measure axial and radial shaft displacements of high speed turbomachinery, a survey of available technology was undertaken. The objective of this survey was to determine the current state-of-the-art for displacement sensors, their capabilities and applicability to the OTV turbomachinery.

A library literature search was made yielding 122 items. The literature covered a wide range of transducer subjects including inductive, capacitive, ultrasonic, optical, and fiberoptic devices. Those that appeared pertinent were obtained and reviewed.

Profiles from 151 companies listed in the "Sensor and Transducer Director" which listed displacement or proximity sensors in their product line were reviewed. Inquiries were sent to 42 companies from this list which appeared to have applicable technology. Most of the responses described devices and technology that were for industrial applications and did not have the accuracy and resolution needed. Examples of some of the commercially available displacement measuring instruments which most nearly meet our requirements are shown on Table A-1. Included in this investigation are the following displacement measuring technologies:

- 1) Inductive
- 2) Capacitive
- 3) Optical
- 4) Ultrasonic

The following assessments were made of the available technologies based on past experience combined with the results of this investigation:

### Inductive

Inductive displacement sensors measure the distance from the sensor to the target by generating a high frequency magnetic field. As the target moves toward the sensor coil eddy currents are generated in the target material which show up as losses in the oscillator bridge network. The variations in losses or distance are

**Table A-1  
Displacement Measurement Instruments**

| Type                   | Manufacturer                             | Range<br>Mil       | Resolution<br>Mil      | Accuracy | Size<br>inches                                  | Temp.<br>Range °F  | Output                    | Freq.<br>Response | Comments  |
|------------------------|--|--------------------|------------------------|----------|---|--------------------|---------------------------|-------------------|---|
| Inductive              | Bently Nevada Corp.<br>Minden, Nevada    | 70                 |                        | ± 1%     | 0.2 dia<br>0.8 Lg                               | -30 -<br>+350      | 200 mv/mil                | 10kHz             | Requires signal condi-<br>tioner (proximitor)<br>Size 2.0x2.4x3.1 in.<br>Larger size probes avail.  |
| Inductive              | Metrix Instrument Co.<br>Houston, TX     | 80                 |                        |          | 0.19 dia<br>1.26 Lg                             | -35°C -<br>+175°C  | 200 mv/mil                | 3kHz              | Requires signal condi-<br>tionr. Used 1mHz excitation<br>to probe.  |
| Inductive              | Kaman Inst. Co.<br>Colorado Springs, CO  | 10                 | 0.004                  | ± 1/2%   | 0.08 dia<br>0.81 Lg<br>#10/32THD                | -65°F -<br>+300°F  | 100 mv/mil                | 50kHz             | Requires signal condi-<br>tioner. Several probe<br>sizes available. Thermal<br>sensitivity shift<br>0.008 mil/°F                              |
| Inductive              | Scientific-Atlanta,<br>Inc., Atlanta, GA | 80                 |                        |          | 0.19 dia<br>0.75 Lg                             | -30 -<br>+350°F    | 200 mv/mil                | 50kHz             | Requires signal condi-<br>tioner  |
| Capacitive             | ADE Corp.<br>Newton, MA                  | 3 to 9             | 1 x 10 <sup>-3</sup>   | ± 0.4%   | 0.10 dia<br>Tip<br>0.280<br>x 1.0 lg<br>overall |                    | ± 10 vdc<br>Full<br>Scale | 40kHz             | Probe contains 4 diodes<br>and is excited from 3kHz<br>driver. External<br>linearization circuits<br>req'd. Min. electrode<br>dia = 0.019 in. |
| Capacitive             | MTI Inc.<br>Latham, N.Y.                 | 0.25 to<br>10      | 10 x 10 <sup>-3</sup>  | ± 0.4%   | .25 x<br>2.5 in.                                | -200°F<br>- +400°F | ± 10 vdc                  | 5kHz              | Special coax cable req'd.<br>No electronics in probe.<br>Higher resolution for<br>smaller range.  |
| Fiberoptic<br>(Lever)  | MTI<br>Latham, N.Y.                      | 0.002 to<br>0.0055 | 1.4 x 10 <sup>-3</sup> | ± 5%     | 0.063 in.<br>x 3.0 in.                          | -100 -<br>300°F    | 6                         | 60kHz             | Probes available to .020"<br>diam. with lesser<br>performance   |
| Fiberoptic<br>(Lever)  | EOTec Corp.<br>West Haven, CT            | ≈.005              | .06 min/mv             |          | .285 dia<br>x .95 in. lg                        |                    | 0-10 volts                | 5kHz              |   |
| Laser<br>Triangulation | DiffRACTO, Ltd.<br>Troy, MI              | 0.080              | 4 x 10 <sup>-2</sup>   | ±1.25%   | 6 x 3.4 x<br>1 in.                              |                    |                           | 200/sec           | 2 inch standoff distance<br>required  |

A-3

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detected and used as a measurement of displacement. A typical block of the sensor system is shown in Figure A-1.

Inductive sensors are widely used for displacement sensing in industrial applications and machinery. ATC has used them successfully in rocket engine turbomachinery for non-cryogenic applications. Sensors are available as small as 0.08 in. dia x 0.8 in. long. Larger diameter sensors measure larger displacements. Output is typically 200 mv per 0.001 inch. Frequency response is as high as 50KHz for some units.

Some of the factors affecting the indicated displacement are:

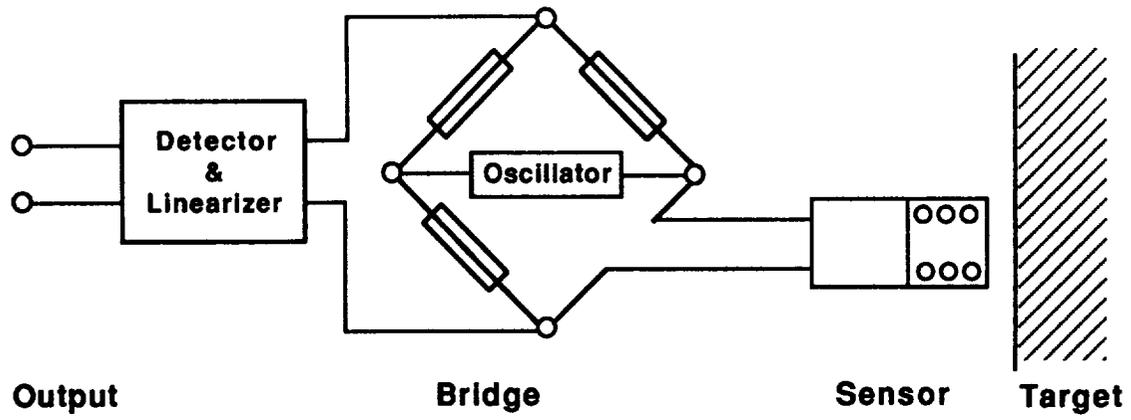
- (a) Target material resistivity and permeability
- (b) Proximity of sensing tip to conducting material
- (c) Temperature of sensor
- (d) Temperature of target

Multiple sensors in close proximity to each other can cause crosstalk and noise which will diminish their accuracy.

Our recent experience in using inductive sensors at liquid nitrogen temperatures (-320°F) has indicated erratic behavior with large signal changes under constant displacement conditions. In another case where sensors were designed specifically for use in liquid hydrogen the performance appears to be better. The initial calibration shows fairly good linearity from 0.005 to 0.0015 inches. Zero shift is from zero to 0.003 inches from temperatures of -320°F to -411°F. To be useful the operating temperature must be known continuously and calibration curves at the appropriate temperature must be available or a temperature compensating signal conditioning unit is needed.

Factors affecting calibration are:

- (a) Coil resistance and temperature coefficient
- (b) Target material resistivity and temperature coefficient
- (c) lead resistance variations



**Figure A-1. Displacement Sensor Schematic**

## Capacitive

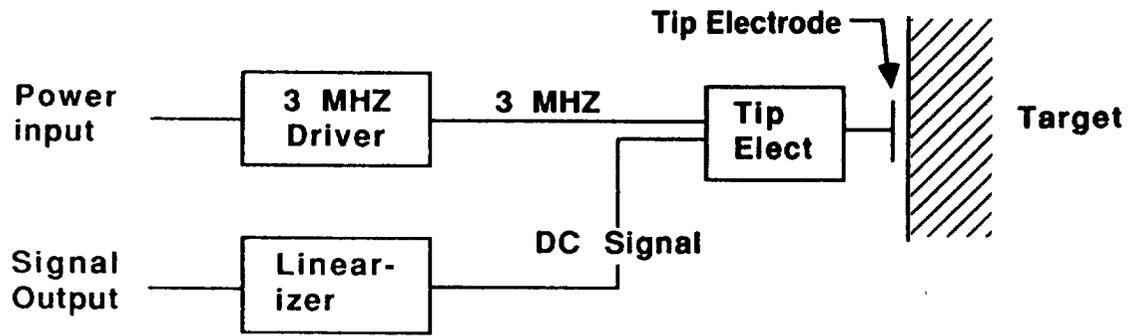
Capacitance displacement sensors use the capacitance between the sensor electrode and the target as a measure of distance. The circuit capacitance is inversely proportional to the distance between the sensor electrode and the target. Because the measured capacitance is very small (in the picofarad range) it is necessary to eliminate the effect of probe and lead leakage capacitance which can be many times the measured capacitance. Several signal conditioning approaches are used to try to meet this objective. Figure A-2 lists the characteristics of two commercially available capacitive instruments capable of precision measurements. The ADE instrument excites the capacitive sensor with a high frequency (3mHz) driver to provide a current proportional to the tip capacitance. The current is rectified by four diodes in the sensor tip to provide a D.C. output for displacement measurement. By placing diodes in the sensor tip leakage capacitance effects are minimized and the output signal is D.C. Frequency response of 50KHz is claimed with resolution better than one microinch for sensor target separation in the range of 0.010 inches. Standard sensor probe diameters as small as 0.280 inches are available. No cryogenic temperature performance is claimed; however, with rectifying diodes as the only temperature sensitive elements in the probe tips performance could be expected to be as good as the available diodes for cryogenic service.

The MTI instrument is an example of another capacitive displacement instrument. This instrument has no active electronics in the sensor tip and uses a guarded circuit to eliminate the effects of stray and cable capacitance. The sensor configuration is shown in Figure A-3.

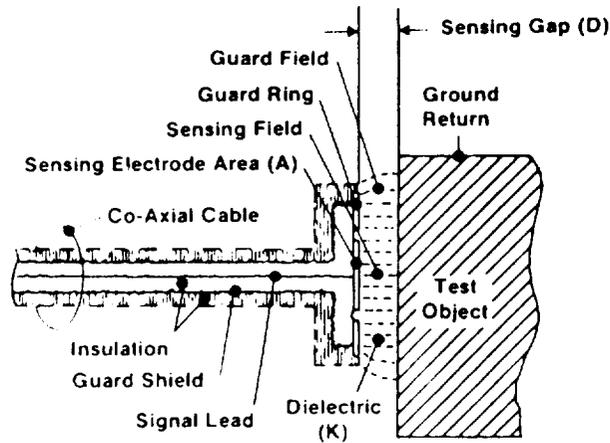
For a range of 0.01 inches its resolution is 10 microinches. Frequency response is 5kHz maximum. Standard sensor diameter is 0.25 in. for a 10 mil range.

## Optical

Optical displacement sensors use light reflected from the target as a means of measuring displacement. To obtain high resolution and accuracy special provisions are required. Some laser-based systems use triangulation to obtain displacement. These systems are usually bulky and not suitable for aerospace applications.



**Figure A-2. Block Diagram of Capacitive Sensor**



**Figure A-3. MTI Sensor Configuration**

One unique system which takes advantage of fiberoptic technology is represented by an instrument manufactured by MTI. It measures light reflected from the target through a bundle of optical fibers. The arrangement is shown in Figure A-4. The sensor system utilizes adjacent pairs of light transmitting and light receiving fibers. The basis for the sensing mechanism is the interaction between the field of illumination of the transmitting fibers and the field of view of the receiving fibers. At contact with the target no light is provided to the receiving fibers. As the gap is increased a rapid increase of light to the receiving fibers is realized until an optical peak is reached. Beyond that gap the light reflected to the receiving fibers decreases according to a square function.

This system provides resolution in the microinch range with sensors as small as 0.063 diameter. Sensors have been made as small as 0.020 diameter with some loss in resolution. Problems due to differential expansion of the fiberoptic bundle and the metal sheath have occurred at cryogenic temperatures. Variations in surface reflectance and media opacity will result in measurement errors. In addition, accurate measurements are difficult to achieve without on-line gap adjustments.

### Ultrasonic

Ultrasonic displacement transducers measure distance by reflecting high frequency acoustic energy from the surface of the target and measuring the transit time. These systems have a resolution capability in the range of 0.001 in. which is inadequate for measuring rotor dynamics of high speed turbomachinery. In addition, media temperature variations can introduce significant errors as sonic speed is a function of temperature.

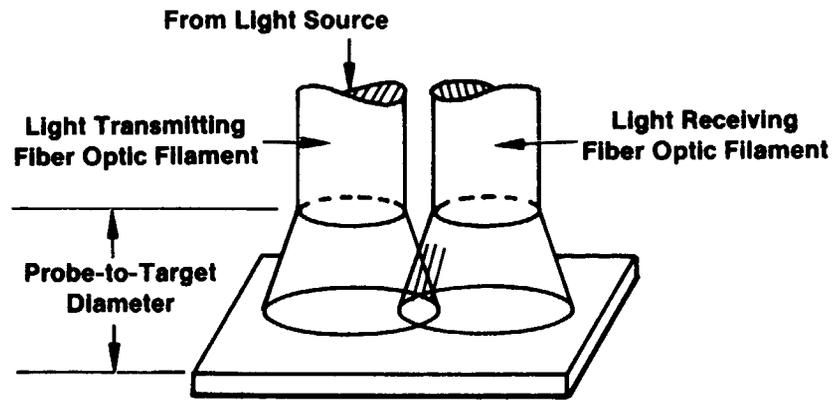
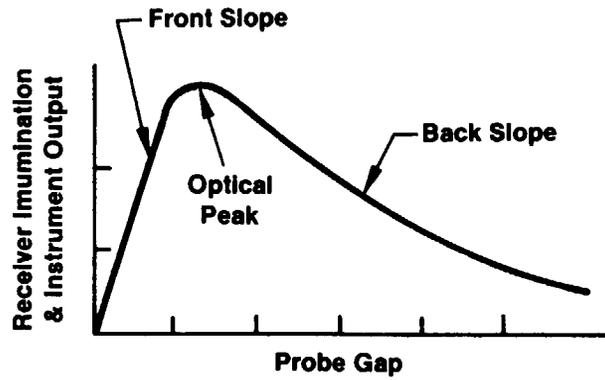
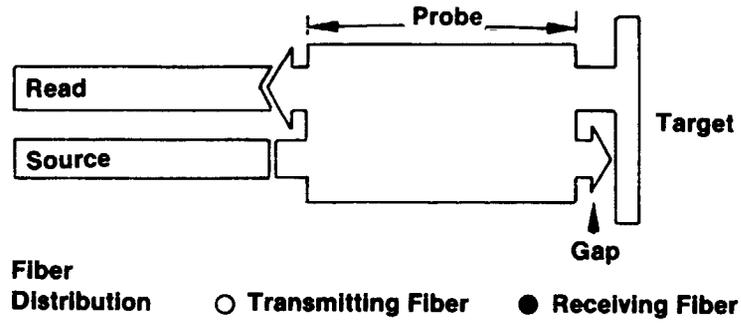
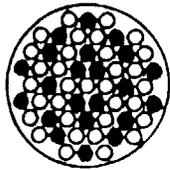


Figure A-4. Fiberoptic Sensor Operation



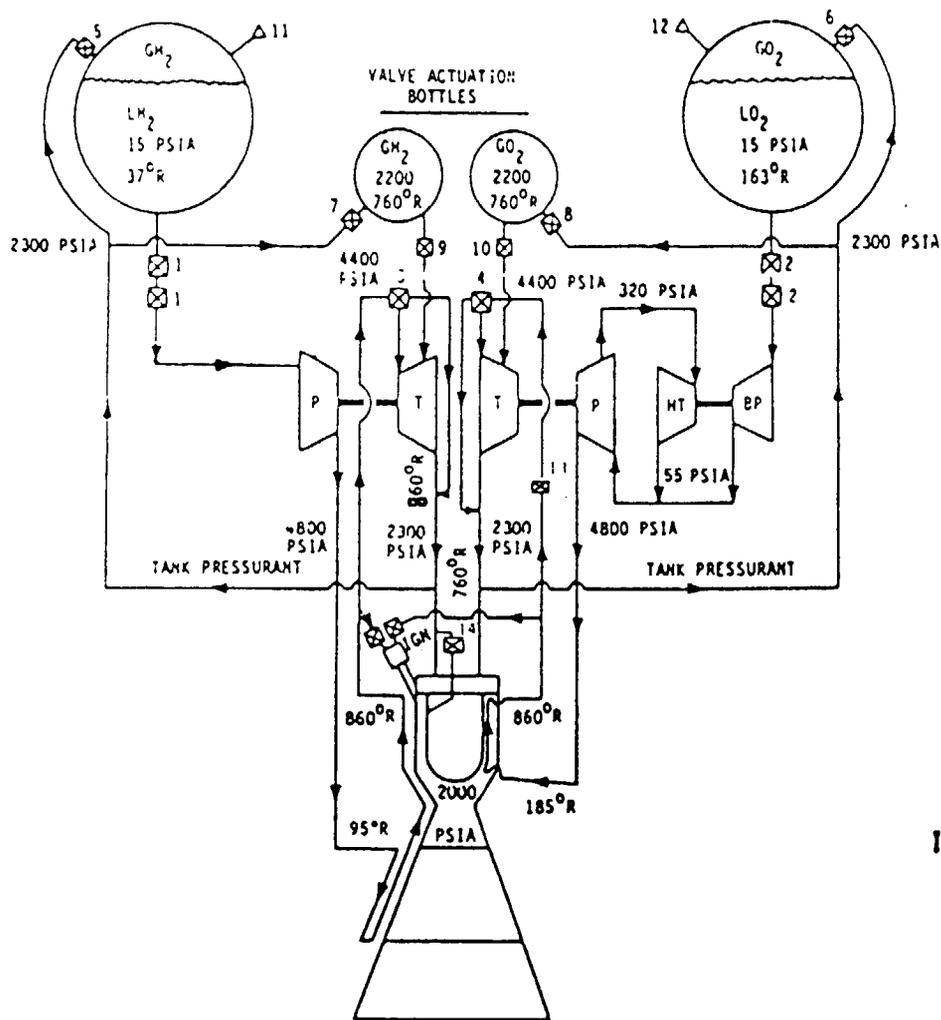
APPENDIX B  
PRELIMINARY VALVE AND SENSOR SPECIFICATIONS

### Preliminary Valve Specifications

This integrated controls and health monitoring project was started while the baseline Aerojet version of the OTV engine was a 3000 lbf thrust dual expander cycle engine. The task was completed after a NASA directed change to the baseline engine thrust was made. The new 7500 lbf thrust engine retained the basic dual expander cycle but engine design activities produced a number of changes to the propellant routing and valves. The original 3K lbf thrust engine schematic is given in Figure B-1. The later version of the 7.5K lbf thrust engine is given in Figure B-2. All specifications for valves are keyed to the requirements of the 7.5K lbf thrust engine. The specification sheets also contain an envelope diagram giving general dimensions for use in layout drawings and initial valve procurement action. These specifications were developed in the engine preliminary design task, and details on criteria and selection will be available in the report for that task. Valve specifications sheets are given as Figure B-4 through B-16.

### Preliminary OTV Engine Sensor Requirements

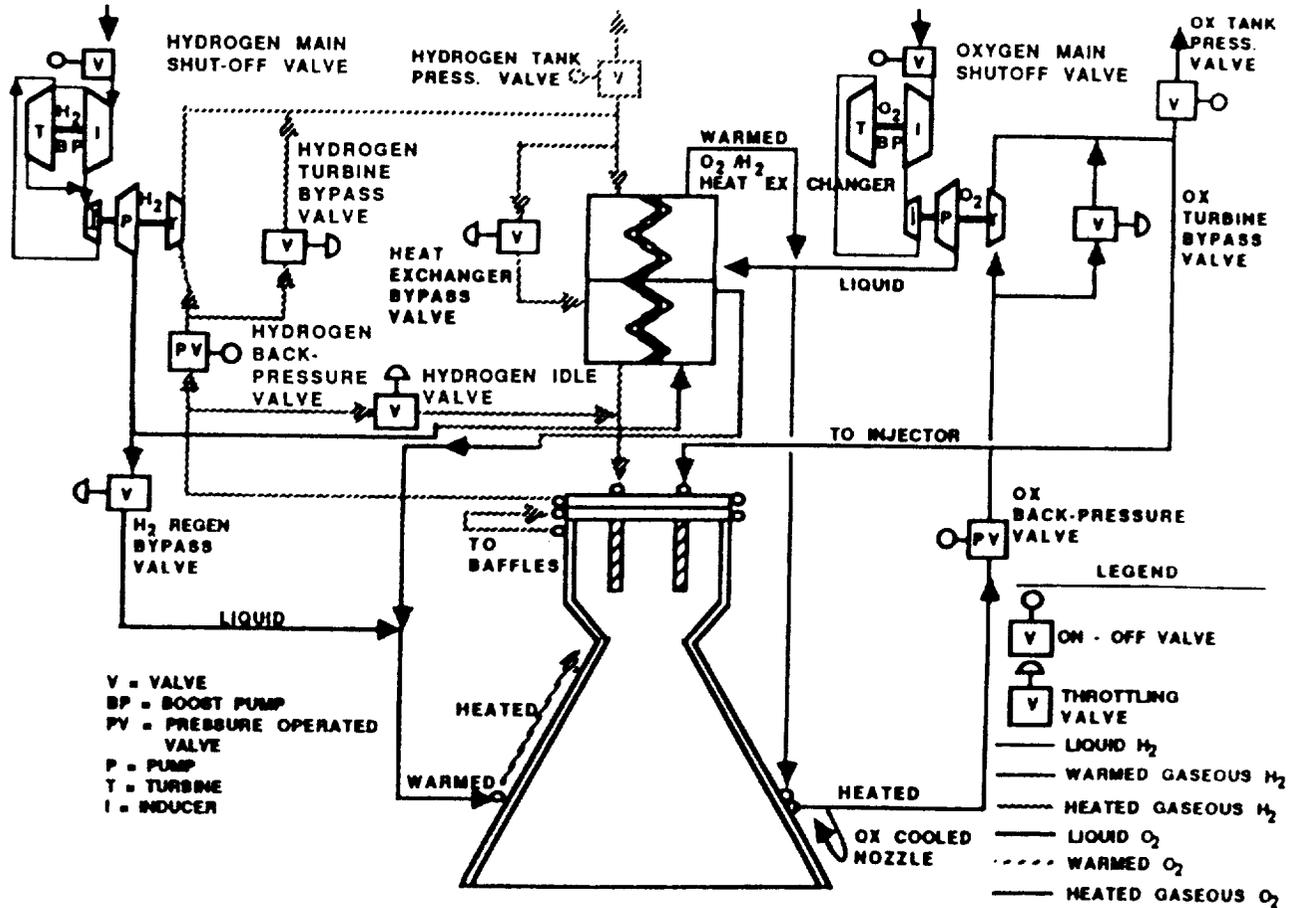
OTV engine sensor requirements were thoroughly reassessed during the engine preliminary design task. Figure B-3 is a schematic of the most recent configuration for the 7500 lbf thrust baseline engine showing sensor locations. This compilation of the sensors proposed for each engine includes those for the nozzle extension and its deployment mechanism and engine compartment sensors not actually installed on the engine but vital to determining safe engine function. The only sensors specific for engine control are the flow meters although the engine controller makes use of a variety of other sensor data. The sensors are divided into two groups: HM-1 for primary functions, and HM-2 for health monitor assessment and backup to the HM-1 group. This list is the result of a pre-screening of candidate sensors, but there was no concerted effort to reduce sensor numbers to any specific figure. Characteristics of the sensors are given in Table B-1. This includes estimates on sensor weight and power requirements. The function of the sensor is given according to priority. When sensor output is used by the engine controller this is the primary function although the data may also be used for health monitoring purposes.



- BP - BOOST PUMP  
 HT - HYDRAULIC TURBINE  
 P - PUMP  
 T - TURBINE
- 1 - LH<sub>2</sub> SHUTOFF VALVES  
 2 - LO<sub>2</sub> SHUTOFF VALVES  
 3 - TURBINE BYPASS VALVE GH<sub>2</sub>  
 4 - TURBINE BYPASS VALVE GO<sub>2</sub>  
 5 - GH<sub>2</sub> PRESSURIZATION CHECK VALVE AND REGULATOR  
 6 - GO<sub>2</sub> PRESSURIZATION CHECK VALVE AND RETULATOR  
 7 - GH<sub>2</sub> START BOTTLE CHECK VALVE  
 8 - GO<sub>2</sub> START BOTTLE CHECK VALVE  
 9 - GH<sub>2</sub> TURBINE START VALVE  
 10 - GO<sub>2</sub> TURBINE START VALVE  
 11 - LH<sub>2</sub> TANK VENT VALVE  
 12 - LO<sub>2</sub> TANK VENT VALVE  
 IGN - IGNITION SYSTEM  
 13 - OXIDIZER BACKPRESSURE VALVE  
 14 - FUEL FILM COOLING VALVE

Figure B-1 Dual Propellant Expander Engine

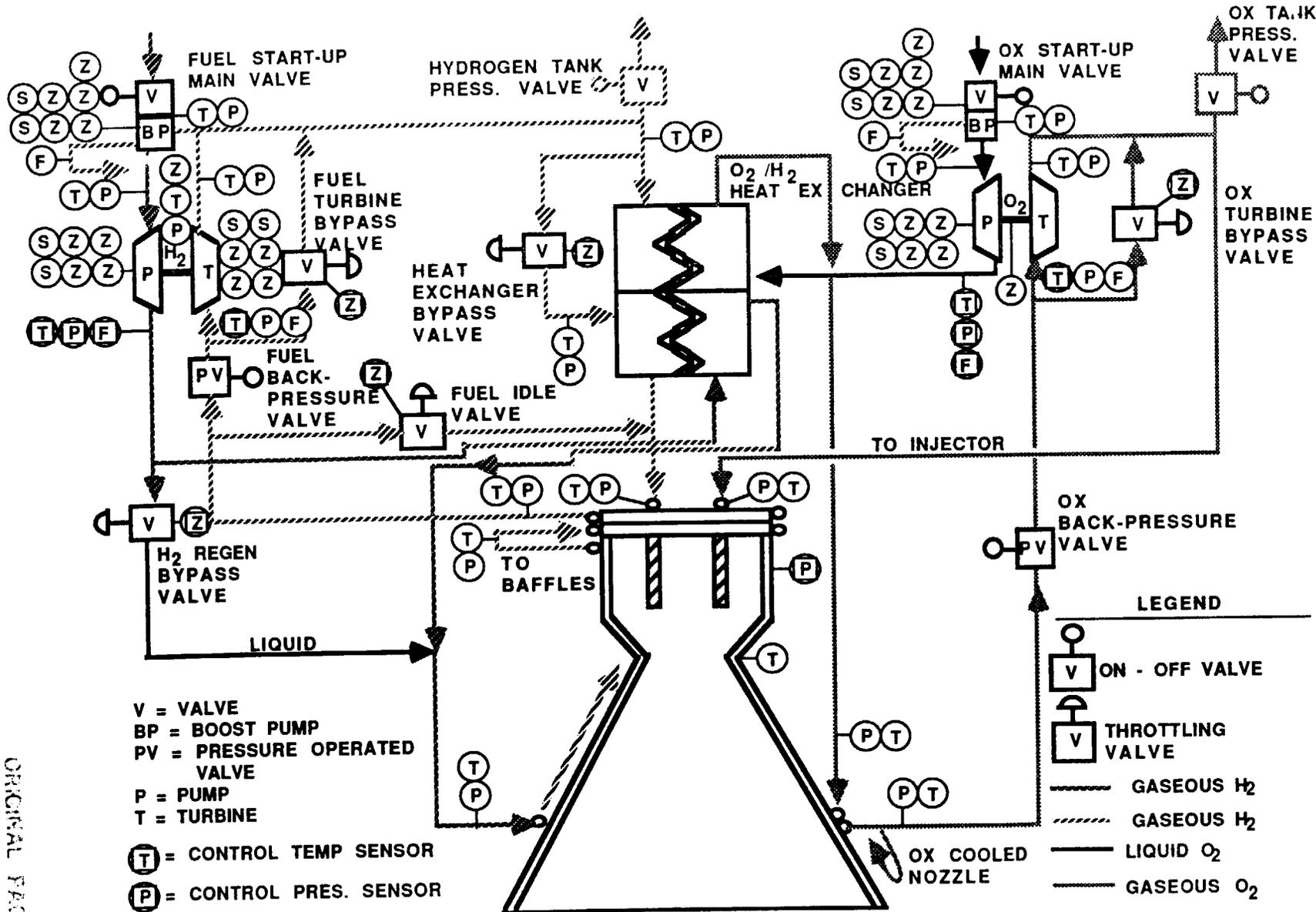
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As of 4 April 1988

Figure B-2 7500 lbs Thrust OTV Engine Dual Expander Cycle

B-5



- V = VALVE
- BP = BOOST PUMP
- PV = PRESSURE OPERATED VALVE
- P = PUMP
- T = TURBINE
- = CONTROL TEMP SENSOR
- = CONTROL PRES. SENSOR
- = CONTROL FLOW SENSOR
- = H-M DISPL. SENSOR
- = H-M SPEED SENSOR

Figure B-3. OTV Engine Dual Expander Cycle Sensor Locations

As of 13 Jan. 1988  
3.1.0.48

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|  |   |
|--|---|
| <b>VALVE FUNCTION :</b>                            | <b>HYDROGEN MAIN SHUT-OFF</b>                     |
| <b>VALVE TYPE :</b>                                | <b>N.C.,ON/OFF,BALL;BFLY;BLADE</b>                |
| <b>ACTUATION METHOD :</b>                          | <b>28 VDC MOTOR</b>                               |
| <b>EST. POWER REQ. :</b>                           | <b>165 WATTS</b>                                  |
| <b>EST. WT. (+-20%):</b>                           | <b>8.5 LBS</b>                                    |
| <b>RESPONSE TIME :</b>                             | <b>250 ms</b>                                     |
| <b>FLUID MEDIA :</b>                               | <b>LH2</b>  |
| <b>OP. PRESS.(psia):</b>                           | <b>0 TO 45</b>                                    |
| <b>OP. TEMP. (F deg):</b>                          | <b>-400 to -422</b>                               |
| <b>DENS.(#/ft**3)(H/L):</b>                        | <b>4.34 / 4.34</b>                                |
| <b>FLOWRATE(#/s)(H/L):</b>                         | <b>2.63 / 0.26</b>                                |
| <b>PRESS. DROP(psi)(H/L) :</b>                     | <b>&lt; 5</b>                                     |
| <b>LEAKAGE ,EXT(sccs of GHe):</b>                  | <b>1 X 10<sup>-4</sup></b>                        |
| <b>MATERIALS OF CONSTRUCTION :</b>                 | <b>ALUMINUM ALLOY</b><br><b>OR TITANIUM ALLOY</b> |
| <b>COMMENTS: FAIL SAFE CLOSED ON LOSS OF POWER</b> |   |
| <b>LINE SIZE :</b>                                 | <b>2.50 O.D.</b>                                  |
| <b>LEAKAGE,INT(sccs of GHe):</b>                   | <b>10 @ 90 psia</b>                               |
| <b>CYCLE LIFE(Min):</b>                            | <b>500</b>  |

B-6

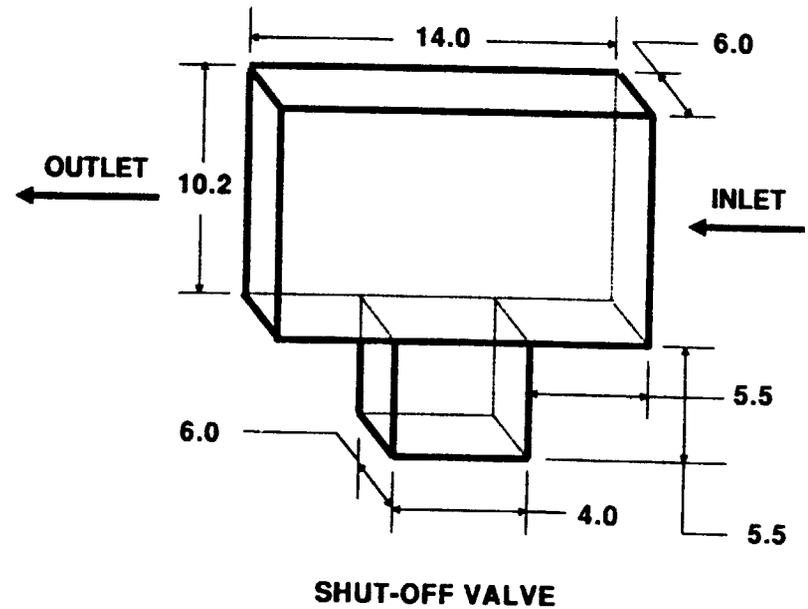


Figure B-4

**VALVE FUNCTION :** OX MAIN SHUT-OFF  
**VALVE TYPE :** N.C.,ON/OFF,BALL;BFLY;BLADE  
**ACTUATION METHOD :** 28 VDC MOTOR  
**EST. POWER REQ. :** 160 WATTS  
**EST. WT. (+-20%):** 8.0 LBS  
**RESPONSE TIME :** 250 ms  
**FLUID MEDIA :** LO2  
**OP. PRESS.(psia):** 0 TO 45 PSIA  
**OP. TEMP. (F deg):** -298  
**DENS.(#/ft\*\*3)(H/L):** 71.2 / 71.2  
**FLOWRATE(#/s)(H/L):** 16.24 / 1.45  
**PRESS. DROP(psi)(H/L) :** < 5  
**LEAKAGE ,EXT(sccs of GHe):** 1 X 10<sup>-4</sup>  
**MATERIALS OF CONSTRUCTION :** ALUMINUM ALLOY  
**COMMENTS :** FAIL SAFE CLOSED ON LOSS OF POWER

**LINE SIZE :** 2.25 O.D.  
**LEAKAGE,INT(sccs of GHe):** 10 @ 90 psia  
**CYCLE LIFE(Min):** 500

B-7

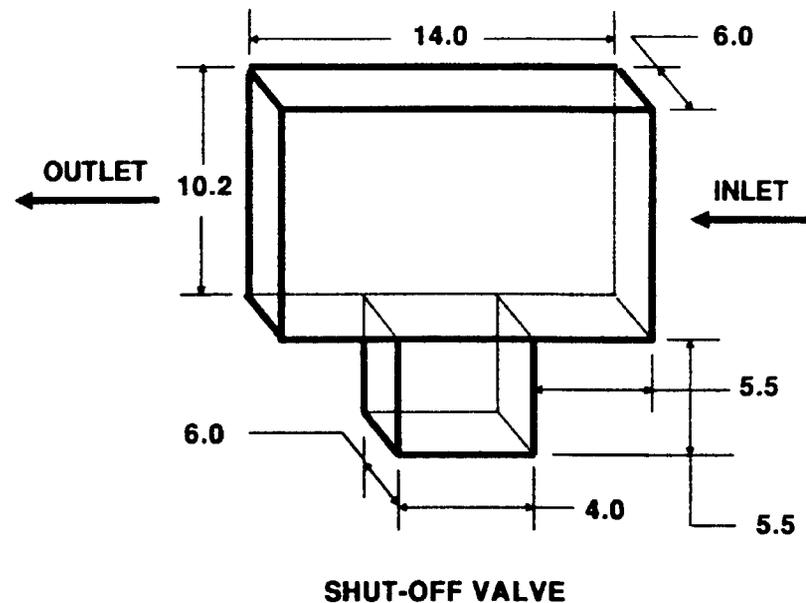


Figure B-5

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|                             |                                       |
|-----------------------------|---------------------------------------|
| VALVE FUNCTION :            | <u>OX TURBINE BYPASS</u>              |
| VALVE TYPE :                | <u>SERVO/PINTLE</u>                   |
| ACTUATION METHOD :          | <u>28 VDC MOTOR</u>                   |
| EST. POWER REQ. :           | <u>60 WATTS</u>                       |
| EST. WT. (+-20%) :          | <u>9.2 LBS</u>                        |
| RESPONSE TIME :             | <u>100 ms</u>                         |
| FLUID MEDIA :               | <u>WARM GO<sub>2</sub></u>            |
| OP. PRESS.(psia)(H/L):      | <u>4937 / 330</u>                     |
| B-8 OP. TEMP. (F deg)(H/L): | <u>400 / 400</u>                      |
| DENS.(#/ft**3)(H/L):        | <u>15.5 / 1.14</u>                    |
| FLOWRATE(#/s)(H/L):         | <u>13.0 / 1.06</u>                    |
| PRESS. DROP(psi)(H/L) :     | <u>2290 / 100</u>                     |
| LEAKAGE ,EXT(sccs of GHe)   | <u>1 x 10<sup>-4</sup></u>            |
| MATERIALS OF CONSTRUCTION : | <u>INCONEL 718 AND</u>                |
| <u>OR MONEL 400</u>         | <u>COMMENT : VALVE FAILS IN PLACE</u> |
| LINE SIZE :                 | <u>.750 O.D.</u>                      |
| OPERATIONAL LIFE(Hrs):      | <u>20</u>                             |

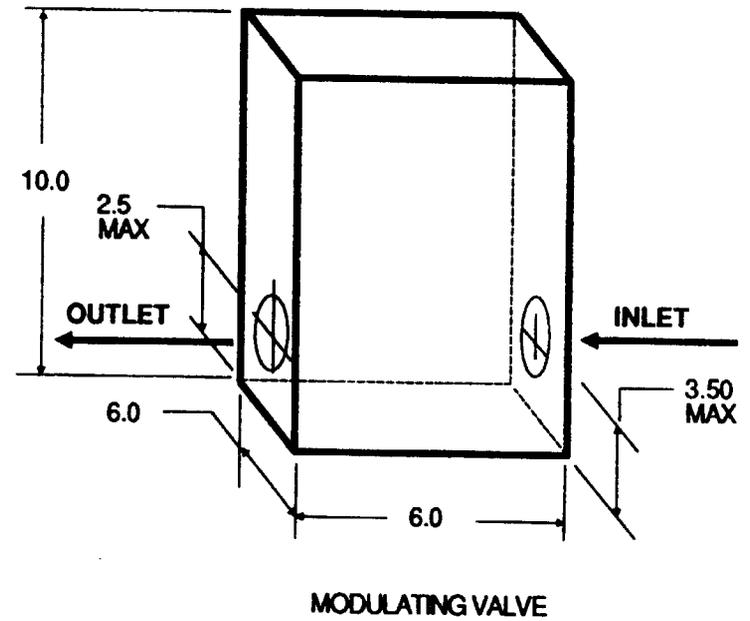


Figure B-6

B-9

|                             |   |
|-----------------------------|---|
| VALVE FUNCTION :            | <u>HYDROGEN TURBINE BYPASS</u>                    |
| VALVE TYPE :                | <u>SERVO/PINTLE</u>                               |
| ACTUATION METHOD :          | <u>28 VDC MOTOR</u>                               |
| EST. POWER REQ. :           | <u>60 WATTS</u>                                   |
| EST. WT. (+-20%) :          | <u>9.2 LBS</u>                                    |
| RESPONSE TIME :             | <u>100 ms</u>                                     |
| FLUID MEDIA :               | <u>HOT GH2</u>                                    |
| OP. PRESS.(psia)(H/L):      | <u>4889 / 298</u>                                 |
| OP. TEMP. (F deg)(H/L):     | <u>540 / 818</u>                                  |
| DENS.(#/ft**3)(H/L):        | <u>0.83 / 0.40</u>                                |
| FLOWRATE(#/s)(H/L):         | <u>0.3 / 0.16</u>                                 |
| PRESS. DROP(psi)(H/L) :     | <u>2210 / 48</u>                                  |
| LEAKAGE EXT.(sccs of GHe):  | <u><math>1 \times 10^{-4}</math></u>              |
| MATERIALS OF CONSTRUCTION : | <u>STAINLESS STEEL OR</u>                         |
|                             | <u>INCONEL 718 COMMENT : VALVE FAILS IN PLACE</u> |
| <hr/>                       |   |
| LINE SIZE :                 | <u>.750 O.D.</u>                                  |
| OPERATIONAL LIFE(Hrs)       | <u>20</u>   |

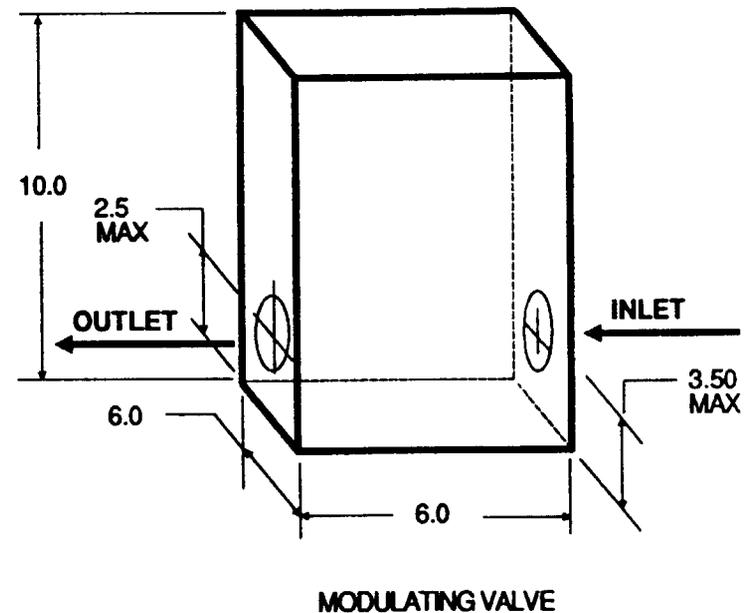


Figure B-7

B-10

|                             |   |
|-----------------------------|---|
| VALVE FUNCTION :            | <u>OX TANK PRESSURIZATION</u>             |
| VALVE TYPE :                | <u>N.C.,ON/OFF,POPPET</u>                 |
| ACTUATION METHOD :          | <u>28 VDC SOLENOID</u>                    |
| EST. POWER REQ. :           | <u>25 WATTS</u>                           |
| EST. WT. (+-20%):           | <u>2.60 LBS</u>                           |
| RESPONSE TIME :             | <u>200 ms</u>                             |
| FLUID MEDIA :               | <u>GO2</u>                                |
| OP. PRESS.(psia):           | <u>0 to 3000</u>                          |
| OP. TEMP. (F deg):          | <u>0 to 400</u>                           |
| DENS.(#/ft**3)(H/L):        | <u>9.77 / .84</u>                         |
| FLOWRATE(#/s)(H/L):         | <u>1.66 / .141</u>                        |
| PRESS. DROP(psi)(H/L) :     | <u>2585 / 215</u>                         |
| LEAKAGE , EXT(sccs of GHe): | <u>1 X 10<sup>-4</sup></u>                |
| MATERIALS OF CONSTRUCTION : | <u>INCONEL 718 OR</u><br><u>MONEL 400</u> |
| LINE SIZE :                 | <u>.375 O.D.</u>                          |
| CYCLE LIFE(Min):            | <u>TBD</u>                                |

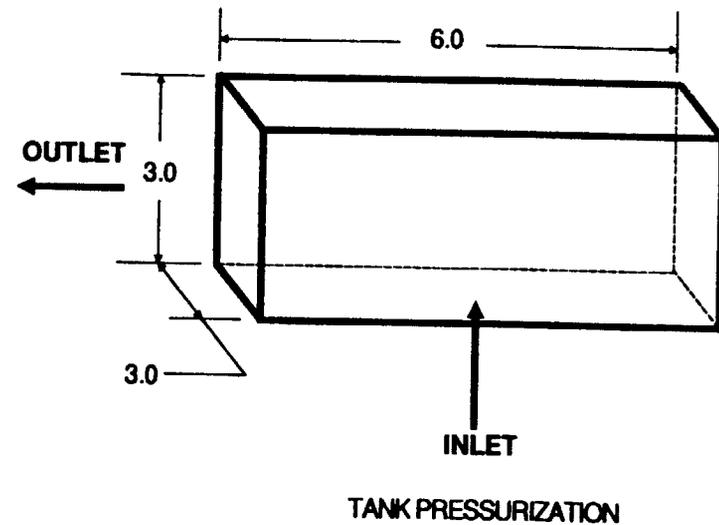


Figure B-8

B-11

|                             | <u>HYDROGEN TANK PRESSURIZATION</u>  |
|-----------------------------|--------------------------------------|
| VALVE FUNCTION :            | <u>N.C.,ON/OFF,POPPET</u>            |
| VALVE TYPE :                | <u>28 VDC SOLENOID</u>               |
| ACTUATION METHOD :          | <u>25 WATTS</u>                      |
| EST. POWER REQ. :           | <u>2.60 LBS</u>                      |
| EST. WT. (+-20%) :          | <u>200 ms</u>                        |
| RESPONSE TIME :             | <u>GH2</u>                           |
| FLUID MEDIA :               | <u>0 to 5000</u>                     |
| OP. PRESS.(psia):           | <u>0 to 800</u>                      |
| OP. TEMP. (F deg):          | <u>0.49 / 0.04</u>                   |
| DENS.(#/ft**3)(H/L):        | <u>.375 / .031</u>                   |
| FLOWRATE(#/s)(H/L):         | <u>2620 / 220</u>                    |
| PRESS. DROP(psi)(H/L) :     | <u><math>1 \times 10^{-4}</math></u> |
| LEAKAGE ,EXT(sccs of GHe):  | <u>STAINLESS STEEL</u>               |
| MATERIALS OF CONSTRUCTION : | <u>OR INCONEL 718</u>                |
| LINE SIZE :                 | <u>.375 O.D.</u>                     |
| CYCLE LIFE(Min):            | <u>TBD</u>                           |

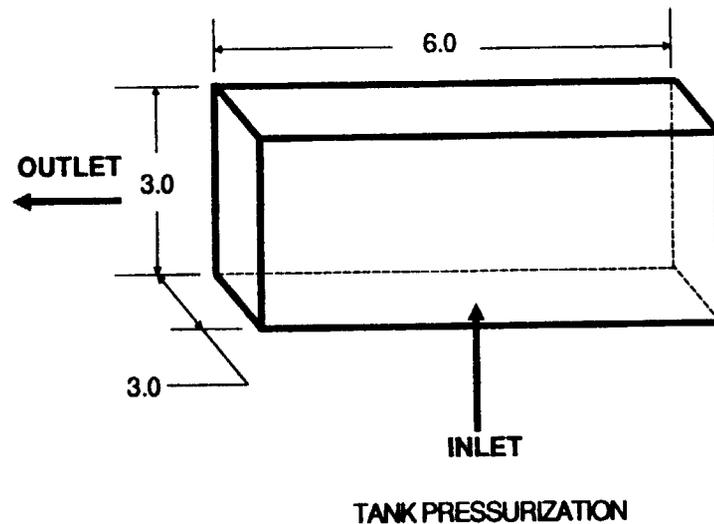


Figure B-9



B-13

|                             |                                  |
|-----------------------------|----------------------------------|
| VALVE FUNCTION :            | <u>OX IGNITER CONTROL</u>        |
| VALVE TYPE :                | <u>N.C., OPEN/CLOSED, POPPET</u> |
| ACTUATION METHOD :          | <u>28 VDC SOLENOID</u>           |
| EST. POWER REQ. :           | <u>24 WATTS</u>                  |
| EST. WT. (+-20%) :          | <u>1 LB</u>                      |
| RESPONSE TIME :             | <u>20 ms</u>                     |
| FLUID MEDIA :               | <u>GO2</u>                       |
| OP. PRESS.(psia):           | <u>0 -6000</u>                   |
| OP. TEMP. (F deg):          | <u>-400 to 100</u>               |
| DENS.(#/ft**3)(H/L):        | <u>4.50/4.49</u>                 |
| FLOWRATE(#/s)(H/L):         | <u>.145 / .0125</u>              |
| PRESS. DROP(psi)(H/L) :     | <u>TBS / TBS</u>                 |
| LEAKAGE ,EXT(sccs of GHe):  | <u>1 X 10<sup>-4</sup></u>       |
| MATERIALS OF CONSTRUCTION : | <u>MONEL 400 OR</u>              |
|                             | <u>INCONEL 718</u>               |
| LINE SIZE :                 | <u>.250 O.D.</u>                 |
| CYCLE LIFE(Min):            | <u>500</u>                       |

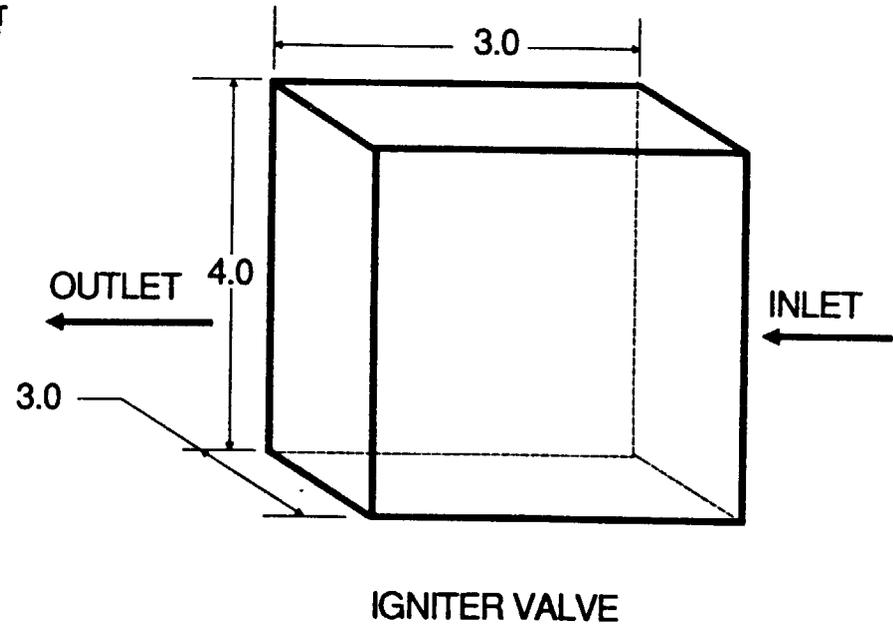
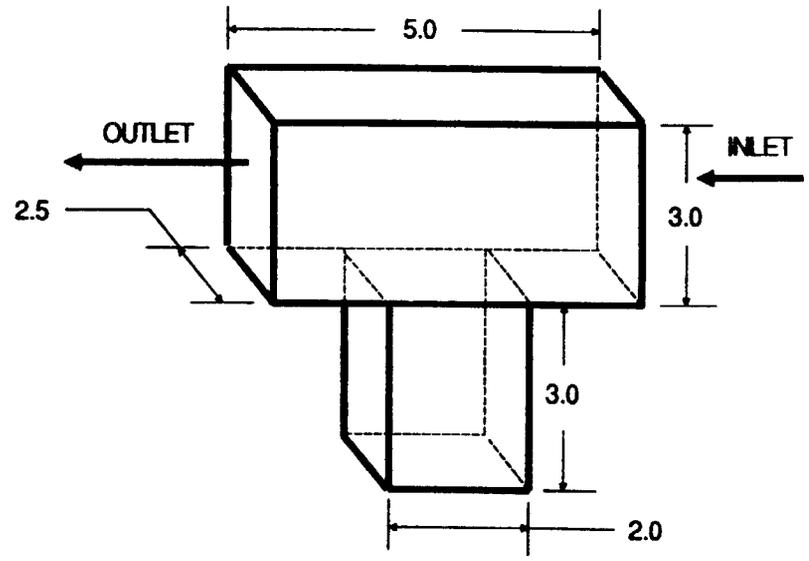


Figure B-11

|                             |   |
|-----------------------------|---|
| VALVE FUNCTION :            | <b>HYDROGEN IDLE</b>                        |
| VALVE TYPE :                | <b>PRESSURE BALANCE, OPEN/PARTIAL CLOSE</b> |
| ACTUATION METHOD* :         | <b>LINE PRESSURE</b>                        |
| EST. POWER REQ. :           | <b>N/A</b>                                  |
| EST. WT. (+-20%) :          | <b>5.0 LBS</b>                              |
| RESPONSE TIME :             | <b>250 ms</b>                               |
| FLUID MEDIA :               | <b>GH2</b>                                  |
| OP. PRESS.(psia):           | <b>80 psid ACTUATE, 0 to 6000 psia</b>      |
| OP. TEMP. (F deg):          | <b>-360 to 540</b>                          |
| DENS.(#/ft**3)(H/L):        | <b>.04</b>                                  |
| FLOWRATE(#/s)(H/L):         | <b>0.0 TO .26</b>                           |
| PRESS. DROP(psi)(H/L) :     | <b>0.0 to 80</b>                            |
| LEAKAGE ,EXT(sccs of GHe):  | <b>1 X 10<sup>-4</sup></b>                  |
| MATERIALS OF CONSTRUCTION : | <b>STAINLESS STEEL</b>                      |

|                        |                  |
|------------------------|------------------|
| LINE SIZE :            | <b>1.00 O.D.</b> |
| OPERATIONAL LIFE(Hrs): | <b>20</b>        |

B-14



HYDROGEN IDLE VALVE

**\*THIS IS AN ALTERNATE VALVE SHOULD MIXTURE RATIO CONTROL ON STARTUP PROVE CRITICAL AND REQUIRE A MODULATING VALVE.**

Figure B-12

B-15

|                             |                               |
|-----------------------------|-------------------------------|
| VALVE FUNCTION :            | <u>HYDROGEN REGEN, BYPASS</u> |
| VALVE TYPE :                | <u>MODULATING POPPET</u>      |
| ACTUATION METHOD :          | <u>28 VDC MOTOR</u>           |
| EST. POWER REQ. :           | <u>60 WATTS</u>               |
| EST. WT. (+-20%) :          | <u>9.2 LBS</u>                |
| RESPONSE TIME :             | <u>100 ms</u>                 |
| FLUID MEDIA :               | <u>H2</u>                     |
| OP. PRESS.(psia)(H/L)       | <u>5580 / 390</u>             |
| OP. TEMP. (F deg)(H/L)      | <u>-340 / -420</u>            |
| DENS.(#/ft**3)(H/L):        | <u>4.50 / 4.49</u>            |
| FLOWRATE(#/s)(H/L):         | <u>0.53 / 0.26</u>            |
| PRESS. DROP(psi)(H/L) :     | <u>75 / 5</u>                 |
| LEAKAGE ,EXT(sccs of GHe)   | <u>1 X 10<sup>-4</sup></u>    |
| MATERIALS OF CONSTRUCTION : | <u>ALUMINUM ALLOY</u>         |
|                             | <u>OR TITANIUM ALLOY</u>      |
| COMMENT :                   | <u>VALVE MUST FAIL OPEN</u>   |
| LINE SIZE :                 | <u>1.00 O.D.</u>              |
| OPERATIONAL LIFE(Hrs):      | <u>20</u>                     |

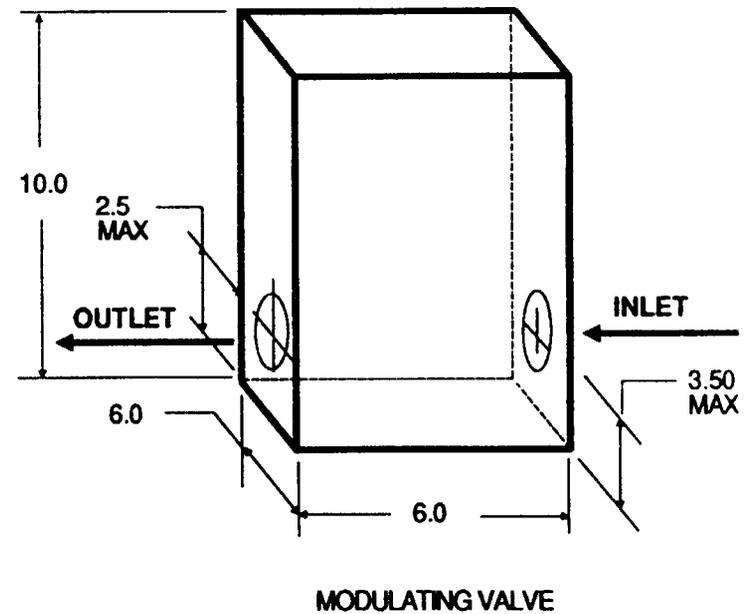
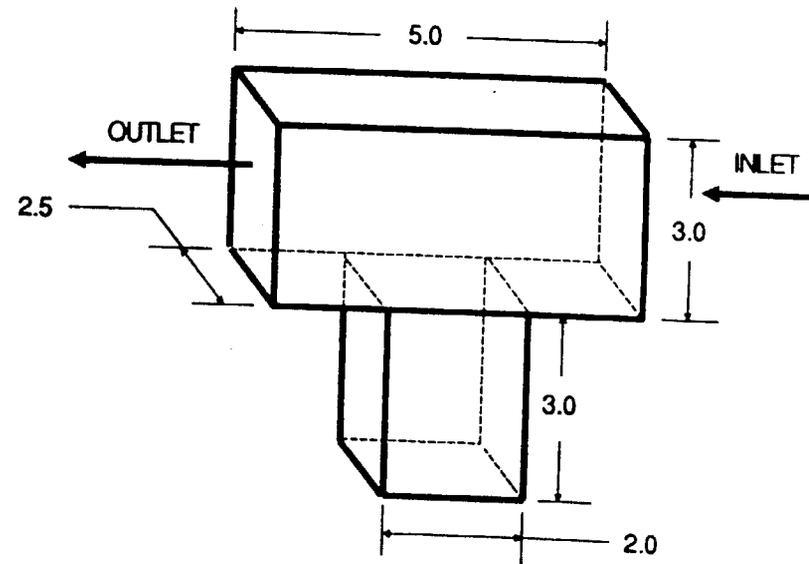


Figure B-13

B-16

|                             |   |
|-----------------------------|---|
| VALVE FUNCTION :            | <u>HYDROGEN IDLE</u>                        |
| VALVE TYPE :                | <u>PRESSURE BALANCE, OPEN/PARTIAL CLOSE</u> |
| ACTUATION METHOD :          | <u>LINE PRESSURE</u>                        |
| EST. POWER REQ. :           | <u>N/A</u>                                  |
| EST. WT. (+-20%) :          | <u>5.0 LBS</u>                              |
| RESPONSE TIME :             | <u>250 ms</u>                               |
| FLUID MEDIA :               | <u>GH2</u>                                  |
| OP. PRESS.(psia):           | <u>80 psid ACTUATE, 0 to 6000 psia</u>      |
| OP. TEMP. (F deg):          | <u>-360 to 540</u>                          |
| DENS.(#/ft**3)(H/L):        | <u>.04</u>                                  |
| FLOWRATE(#/s)(H/L):         | <u>0.0 TO .26</u>                           |
| PRESS. DROP(psi)(H/L) :     | <u>0.0 to 80</u>                            |
| LEAKAGE ,EXT(sccs of GHe):  | <u>1 X 10<sup>-4</sup></u>                  |
| MATERIALS OF CONSTRUCTION : | <u>STAINLESS STEEL</u>                      |
| <hr/>                       |   |
| LINE SIZE :                 | <u>1.00 O.D.</u>                            |
| OPERATIONAL LIFE(Hrs):      | <u>20</u>                                   |



HYDROGEN IDLE VALVE

Figure B-14

B-17

|                             |  |
|-----------------------------|--|
| VALVE FUNCTION :            | <u>HYDROGEN BACK PRESSURE</u>            |
| VALVE TYPE :                | <u>N.C., OPEN/PARTIAL CLOSED, POPPET</u> |
| ACTUATION METHOD :          | <u>LINE PRESSURE</u>                     |
| EST. POWER REQ. :           | <u>N/A</u>                               |
| EST. WT. (+-20%) :          | <u>3.0 LBS</u>                           |
| RESPONSE TIME :             | <u>250 ms @ 500 psia</u>                 |
| FLUID MEDIA :               | <u>GH2</u>                               |
| OP. PRESS.(psia):           | <u>0 to 4889</u>                         |
| OP. TEMP. (F deg):          | <u>50 to 818</u>                         |
| DENS.(#/ft**3)(H/L):        | <u>0.83 / 0.04</u>                       |
| FLOWRATE(#/s)(H/L):         | <u>2.63 / 0.26</u>                       |
| PRESS. DROP(psi)(H/L) :     | <u>10.0 / 2.2</u>                        |
| LEAKAGE ,EXT(sccs of GHe):  | <u>1 x 10<sup>-4</sup></u>               |
| MATERIALS OF CONSTRUCTION : | <u>INCONEL 718 OR</u>                    |
|                             | <u>STAINLESS STEEL</u>                   |
| LINE SIZE :                 | <u>1.25 O.D.</u>                         |
| CYCLE LIFE(Min):            | <u>500</u>                               |

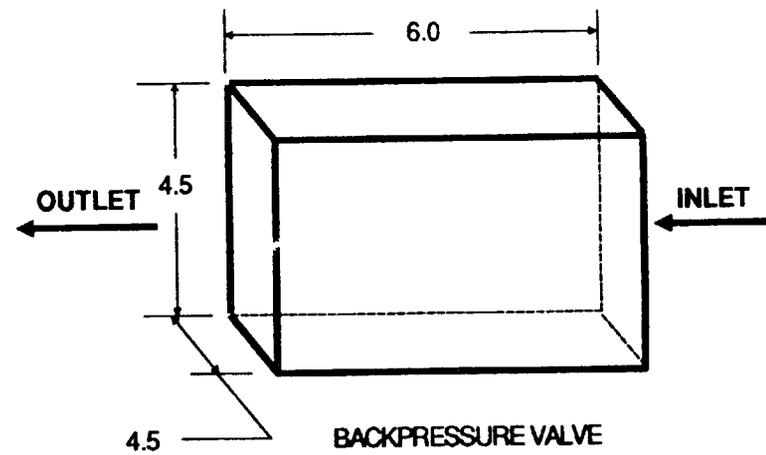


Figure B-15

B-18

|                             |   |
|-----------------------------|---|
| VALVE FUNCTION :            | <u>OX BACKPRESSURE</u>                                  |
| VALVE TYPE :                | <u>N.C., <small>OPEN/PARTIAL</small> CLOSED, POPPET</u> |
| ACTUATION METHOD :          | <u>LINE PRESSURE</u>                                    |
| EST. POWER REQ. :           | <u>N/A</u>  |
| EST. WT. (+-20%) :          | <u>5.0 LBS</u>  |
| RESPONSE TIME :             | <u>250 ms @ 500psia</u>                                 |
| FLUID MEDIA :               | <u>GO2</u>  |
| OP. PRESS.(psia):           | <u>0 to 4937</u>  |
| OP. TEMP. (F deg):          | <u>-260 to 400</u>                                      |
| DENS.(#/ft**3)(H/L):        | <u>15.5 / 1.14</u>                                      |
| FLOWRATE(#/s)(H/L):         | <u>16.24 / 1.45</u>                                     |
| PRESS. DROP(psi)(H/L) :     | <u>22.0 / 2.4</u>                                       |
| LEAKAGE ,EXT(sccs of GHe):  | <u>1 x 10<sup>-4</sup></u>                              |
| MATERIALS OF CONSTRUCTION : | <u>INCONEL 718 OR</u><br><u>MONEL 400</u>               |
| LINE SIZE :                 | <u>1.75 O.D.</u>  |
| CYCLE LIFE(Min):            | <u>500</u>  |

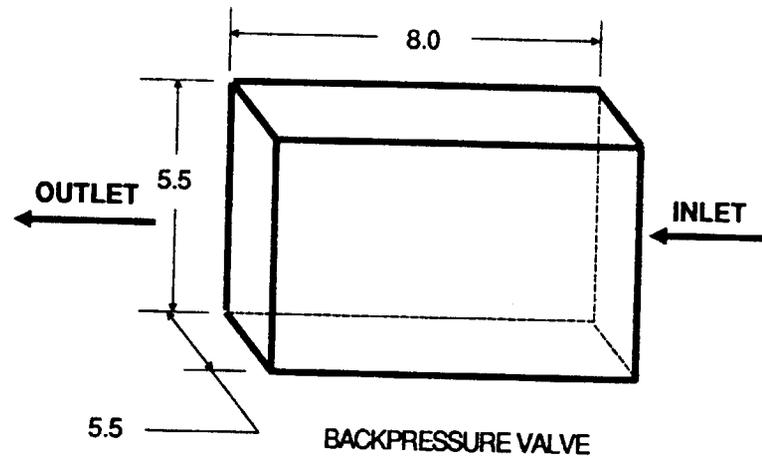


Figure B-16

**Table B-1.  
Health Monitor and Control Sensor List**

| Component                | Sensor                           | Location  | Function |      |      | Wt.                     | PWR.                           |                |
|--------------------------|----------------------------------|---|----------|------|------|-------------------------|--------------------------------|----------------|
|                          |                                  |   | Control  | HM-1 | HM-2 |                         |                                |                |
| Hydrogen Turbopump Assy. | Z1 Shaft Axial Displ.            | Inside First Stage Turbo-pump Assy.             |          | X    |      | 2.5 for 3 Functions     | 10 VDC<br>30MA for 3 Functions |                |
|                          | Z2 Shaft Radial Displ.           | ↓   |          | X    |      |                         |                                |                |
|                          | S1 Speed                         | ↓   |          | X    |      |                         |                                |                |
|                          | Z3 Shaft Axial Displ.            | Inside First Stage Turbo-pump Assy. 90° from Z1 |          | X    |      | 2.5 for 3 Functions     | 10 VDC<br>30MA for 3 Functions |                |
|                          | Z4 Shaft Radial Displ.           | ↓   |          | X    |      |                         |                                |                |
|                          | S2 Speed                         | ↓   |          |      | X    |                         |                                |                |
|                          | P-1 Pump Discharge Pres.         | To Line @ Pump Disch.                           |          | XX   |      | 0.5 Oz.                 | 10 VDC<br>15MA                 |                |
|                          | T-1 Pump Discharge Temp.         | To Line @ Pump Disch.                           |          | XX   |      | 0.7 Oz.                 | 10 VDC<br>15MA                 |                |
|                          | F-1 Pump Flow                    | ↓   |          | XX   |      | 12 Oz.                  | 28 VDC<br>20MA                 |                |
|                          | Z5 TPA Vibration                 | On T-P Housing Between Brgs.                    |          |      | X    | .07 Oz.                 | 15 VDC<br>4MA                  |                |
| Hydrogen Turbopump Assy. | Z6 Shaft Axial Displ.            | Inside 2nd Stage Turbo-pump Assy.               |          | X    |      | 2.5 Oz. for 3 Functions | 10 VDC<br>30MA for 3 Functions |                |
|                          | Z7 Shaft Radial Displ.           | ↓   |          | X    |      |                         |                                |                |
|                          | S3 Speed                         | ↓   |          | X    |      |                         |                                |                |
|                          | Z8 Shaft Axial Displ.            | Inside 2nd Stage Turbo-pump Assy. 90° from Z3   |          |      | X    | 2.5 Oz. for 3 Functions | 10 VDC<br>30MA for 3 Functions |                |
|                          | Z9 Shaft Radial Displ.           | ↓   |          |      | X    |                         |                                |                |
|                          | S4 Speed                         | ↓   |          |      | X    |                         |                                |                |
|                          | P2 Pump Interstage Pres.         | To TPA Between Stages                           |          |      |      | X                       | 0.5 Oz.                        | 10 VDC<br>15MA |
|                          | T2 Pump Interstage Temp.         | On TPA Between Stages                           |          |      |      | X                       | 0.7 Oz.                        | 10 VDC<br>15MA |
|                          | P3 Turbine 2nd Stage Inlet Pres. | On TP Housing Between Stages                    |          |      |      | X                       | 0.5 Oz.                        | 10 VDC<br>15MA |
|                          | T3 Turbine 2nd Stage Inlet Temp. | ↓   |          |      |      | X                       | 0.3 Oz.                        | 10 VDC<br>15MA |

**Table B-1.  
Health Monitor and Control Sensor List (Cont.)**

| Component                      | Sensor                               | Location   | Function               |      |      | Wt.                        | PWR.                              |
|--------------------------------|--------------------------------------|--|------------------------|------|------|----------------------------|-----------------------------------|
|                                |                                      |  | Control                | HM-1 | HM-2 |                            |                                   |
| Hydrogen Turbopump Assy.       | P4 Pump Inlet Pres.                  | In Line @ Pump Inlet                                 |                        |      | X    | 0.5 Oz.                    | 10 VDC<br>15MA                    |
|                                | P5 Turbine Inlet Pres.               | In Line @ Turbine Inlet                              |                        | X    |      |                            |                                   |
|                                | P6 Turbine Disch Pres.               | In Line @ Turbine Outlet                             |                        |      | X    |                            |                                   |
|                                | T4 Turbine Inlet Temp.               | In Line @ Turbine Inlet                              | XX                     |      |      | 0.7 Oz.                    | 10 VDC<br>15MA                    |
|                                | T5 Turbine Disch. Temp               | In Line @ Turbine Disch.                             |                        |      | X    | 0.7 Oz.                    | 10 VDC<br>15MA                    |
|                                | F2 Turbine Inlet Flow                | In Line @ Turbine Inlet                              |                        |      | X    | 12 Oz.                     | 28 VDC<br>20MA                    |
|                                | T6 Pump Inlet Temp.                  | In Line @ Pump Inlet                                 |                        |      | X    | 0.7 Oz.                    | 10 VDC<br>15MA                    |
| Hydrogen Boost Turbopump Assy. | Z10 Bearing Outer Race Deflectometer | On Bearing Outer Race<br>.005 In. Nominal Clearances |                        | X    |      | 0.8 Oz.                    | 10V<br>20MA for<br>2 Sensors      |
|                                | Z11 Bearing Outer Race Deflectometer |  |                        | X    |      | 0.8 Oz.                    | 2 Sensors                         |
|                                | Z12 Boost Pump Accel.                | On Boost Pump Housing Between Brgs.                  |                        |      | X    | 0.07 Oz.                   | 15 VDC<br>4MA                     |
|                                | P7 Boost Pump Inlet Pres.            | In Line @ Pump Inlet                                 |                        |      | X    | 0.5 Oz.                    | 10 VDC<br>15MA                    |
|                                | T-7 Boost Pump Inlet Temp.           | In Line @ Pump Inlet                                 |                        |      | X    | 0.7 Oz.                    | 10 VDC<br>15MA                    |
|                                | F3 Boost Turbine Inlet Flow          | In Line @ Turbine Inlet                              |                        |      | X    | 12 Oz.                     | 28 VDC<br>20MA                    |
|                                | Oxygen Turbopump Assy.               | Z13 Shaft Axial Displ.                               | Inside Turbopump Assy. |      | X    |                            | 2.5 Oz. for<br>3 Functions        |
| Z14 Shaft Radial Displ.        |                                      |  |                        | X    |      |                            |                                   |
| S5 Speed                       |                                      |  |                        | X    |      |                            |                                   |
| Z15 Shaft Axial Displ.         |                                      | Inside Turbopump Assy.<br>90° from Z5                |                        |      | X    | 2.5 Oz. for<br>3 Functions | 10 VDC<br>30MA for<br>3 Functions |
| Z16 Shaft Radial Displ.        |                                      |  |                        | X    |      |                            |                                   |
| S6 Speed                       |                                      |  |                        |      | X    |                            |                                   |

**Table B-1.  
Health Monitor and Control Sensor List (Cont.)**

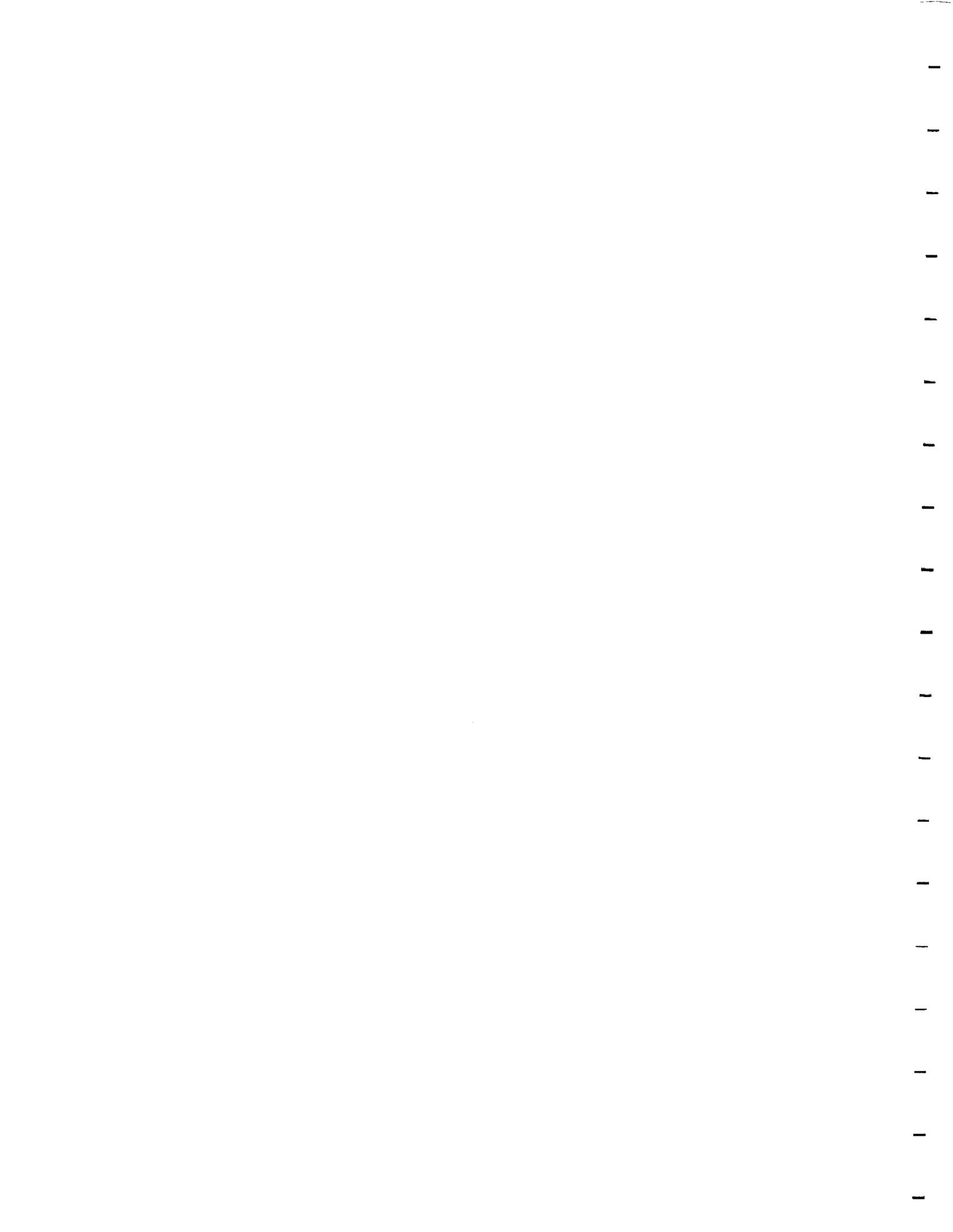
| Component                     | Sensor                               | Location  | Function |      |      | Wt.      | PWR.                            |
|-------------------------------|--------------------------------------|---|----------|------|------|----------|---------------------------------|
|                               |                                      |   | Control  | HM-1 | HM-2 |          |                                 |
| Oxygen Turbopump Assy.        | P8 Pump Disch. Pres.                 | In Line @ Pump Disch.   | XX       |      |      | 0.5 Oz.  | 10 VDC<br>15MA                  |
|                               | T7 Pump Disch. Temp.                 | In Line @ Pump Disch.   | XX       |      |      | 0.7 Oz.  | 10 VDC<br>15MA                  |
|                               | F4 Pump Flow                         | ↓   | XX       |      |      | 12 Oz.   | 28 VDC<br>20MA                  |
| Oxygen Turbopump Assy.        | Z17                                  | On T.P. Housing Between Bearings                              |          | X    |      | 0.07 Oz. | 15 VDC<br>4MA                   |
|                               | P9 Pump Inlet Pres.                  | In Line @ Pump Inlet  |          |      | X    | 0.5 Oz.  | 10 VDC<br>15MA                  |
|                               | P10 Turbine Inlet Pres.              | In Line @ Turbine Inlet                                       |          | X    |      | ↓        | ↓                               |
|                               | T8 Turbine Inlet Temp.               | In Line @ Turbine Inlet                                       | XX       |      |      | 0.7 Oz.  | 10 VDC<br>15MA                  |
|                               | T9 Turbine Disch. Temp.              | In Line @ Turbine Disch.                                      |          |      | X    | 0.7 Oz.  | ↓                               |
|                               | T10 Pump Inlet Temp.                 | In Line @ Pump Inlet  |          |      | X    | 0.7 Oz.  | ↓                               |
|                               | F5 Turbine Inlet Flow                | In Line @ Turbine Inlet                                       |          |      | X    | 12 Oz.   | 28 VDC<br>20MA                  |
|                               | Z18 Bearing Outer Race Deflectometer | On Bearing Outer Race<br>.005 in. Nominal Clearance           |          | X    |      | 0.8 Oz.  | 28 VDC<br>20MA for<br>2 Sensors |
|                               | Z19 Bearing Outer Race Deflectometer | ↓   |          | X    |      | ↓        | ↓                               |
|                               | Z20 Boost Pump Accel.                | On Boost Pump Hsg. Between Brgs.                              |          |      | X    | .07 Oz.  | 15 VDC<br>4MA                   |
| Oxygen Boost Turbo-pump Assy. | P11 Boost Pump Inlet Pres.           | In Line @ Pump Inlet<br>Thermal Isolation Req'd by Tube Conn. |          |      | X    | 0.5 Oz.  | 10 VDC<br>15MA                  |
|                               | T11 Boost Pump Inlet Temp.           | In Line @ Pump Inlet  |          |      | X    | .7 Oz.   | 10 VDC<br>15MA                  |
|                               | F6 Boost Turbine Inlet Flow          | In Line @ Turbine Inlet                                       |          |      | X    | .12 Oz.  | 28 VDC<br>20MA                  |
|                               | P12 Chamber Pressure                 | Combustion Chamber  | XX       |      |      | 0.5 Oz.  | 10 VDC<br>15MA                  |
| Combustion Chamber            |                                      |   |          |      |      |          |                                 |

**Table B-1.  
Health Monitor and Control Sensor List (Cont.)**

| Component  | Sensor                                  | Location                                     | Function |      |      | Wt.     | PWR.                |
|--|---|--|----------|------|------|---------|---------------------|
|  |   |  | Control  | HM-1 | HM-2 |         |                     |
| Injector<br>↓<br>Combustion Chamber<br>↓<br>Nozzle<br>↓<br>Regen | P13 Fuel Inlet Pressure                 | At Injector Fuel Inlet                       |          | X    |      | 0.3 Oz. | 10 VDC<br>15MA<br>↓ |
|  | T12 Fuel Inlet Temp.                    | ↓  |          | X    |      | 0.7 Oz. |                     |
|  | P14 Oxidizer Inlet Pressure             | At Injector Oxidizer Inlet                   |          | X    |      | 0.5 Oz. |                     |
|  | T13 Oxidizer Inlet Temp.                | ↓  |          | X    |      | 0.7 Oz. |                     |
|  | P15 Chamber Coolant Outlet Pres.        | At Chamber Coolant Outlet                    |          | X    |      | 0.5 Oz. |                     |
|  | T14 Chamber Coolant Outlet Temp.        | ↓  |          | X    |      | 0.7 Oz. |                     |
|  | P16 Chamber Baffle Coolant Outlet Pres. | At Transition Between Chamber Baffle Coolant |          |      | X    | 0.5 Oz. |                     |
|  | T15 Chamber Baffle Coolant Outlet Temp. | ↓  |          |      | X    | 0.7 Oz. |                     |
|  | T16 Chamber Throat Surface Temp.        | Multiple Sensor Band Around Chamber Throat   |          |      | X    | TBD     |                     |
|  |   |  |          |      |      |         |                     |
|  | P17 Fuel Coolant Inlet Pres.            | At Nozzle Fuel Inlet Manifold                |          | X    |      | 0.5 Oz. | 10 VDC<br>15MA<br>↓ |
|  | T17 Fuel Coolant Inlet Temp.            | ↓  |          | X    |      | 0.7 Oz. |                     |
|  | P18 Ox. Coolant Inlet Pres.             | At Nozzle Ox. Inlet Manifold                 |          | X    |      | 0.5 Oz. |                     |
|  | T18 Ox. Coolant Inlet Temp.             | ↓  |          | X    |      | 0.7 Oz. |                     |
|  | P19 Ox. Coolant Outlet Pres.            | At Nozzle Ox. Outlet Manifold                |          |      | X    | 0.5 Oz. |                     |
|  | T19 Ox. Coolant Outlet Temp.            | ↓  |          |      | X    | 0.7 Oz. |                     |
|  | T20 Fuel Regen Outlet Temp.             | @ Fuel Regen Sec. Outlet Port.               |          | X    |      | 0.7 Oz. |                     |
|  | T21 Ox. Regen Fuel Inlet Temp.          | Ox. Regen Inlet Port                         |          |      | X    | 0.7 Oz. |                     |
|  | P20 Fuel Regen Inlet Pres.              | Fuel Regen Inlet Port                        |          |      | X    | 0.5 Oz. |                     |

**Table B-1.  
Health Monitor and Control Sensor List (Cont.)**

| Component                 | Sensor                                 | Location                       | Function                           |      |      | Wt.                      | PWR.        |
|---------------------------|--|--------------------------------|------------------------------------|------|------|--------------------------|-------------|
|                           |  |                                | Control                            | HM-1 | HM-2 |                          |             |
| Fuel Turbine Bypass Valve | Z21 Position                           | On Valve Pintle Extension      | XX                                 |      |      | 3.2 Oz. for 1 in. Stroke | 28 VDC 20MA |
| Ox. Turbine Bypass Valve  | Z22 Position                           | ↓                              | XX                                 |      |      | ↓                        | ↓           |
| Hex. Bypass Valve         | Z23 Position                           |                                | XX                                 |      |      |                          |             |
| Fuel Idle Valve           | Z24 Position                           |                                | XX                                 |      |      |                          |             |
| Fuel Regen, Bypass Valve  | Z25 Position                           |                                | XX                                 |      |      |                          |             |
| Nozzle Position           | Z26 Position-Ret.<br>Z27 Position-Ext. |                                | Limit Switches at Limits of Travel | XX   |      |                          |             |
| Engine Compartment        | T22, T23, T24, T25 Temperature         | In Engine Compartment 4 Places |                                    | X    |      | 0.02 Oz.                 | 10 VDC 15MA |
| Engine Compartment        | L1, L2, L3, L4 H2 Leak Detection       | In Engine Compartment 4 Places |                                    | X    |      | TBD                      | TBD         |
| Engine Compartment        | L5, L6, L7, L8 O2 Leak Detection       | In Engine Compartment 4 Places |                                    | X    |      | ↓                        | ↓           |
| <b>Total</b>              |  |                                | 16 + 16                            | 46   | 31   |                          |             |



APPENDIX C  
OTV ENGINE FAILURE MODES AND EFFECTS ANALYSIS

OTV DUAL PROPELLANT EXPANDER CYCLE ENGINE  
FAILURE MODES AND EFFECTS ANALYSIS

| COMPONENT OR SUBSYSTEM  | FAILURE MODE   | SYSTEM EFFECT   | SYSTEM FAILURE |         |
|---|--|---|----------------|---------|
|   |  |   | FULL           | PARTIAL |
| Propellant Inlet Shutoff Valves (LO <sub>2</sub> /LH <sub>2</sub> )<br><br>(Series redundant/one valve on each side of interface) | 1. 1 unit fails to close.  | 1. No effect - series redundant.  |                |         |
|   | 2. Both units fail to close.   | 2. No effect - close tank isolation valve.  |                | X       |
|   | 3. 1 or 2 unit(s) fail to open.  | 3. No engine operation W/O EVA and manual operation of valve.   | X              |         |
|   | 4. 1 unit fails partially open or closed.  | 4. Engine may operate at partial thrust but can still be throttled, started, shutdown.  |                | X       |
|   | 5. Response time out of tolerance  | 5. No effect on closing (series redundant). No effect on opening due to to (relatively) long start time operation (tank head idle mode).  |                |         |
| Oxidizer Turbopump Assembly   | 1. Loss of LO <sub>2</sub> to hydrostatic bearings (contamination).  | 1. Bearings have redundant lubrication passages - partial loss may result in bearing wear. Significant or total loss may cause ignition or melting and HM directed shutdown. (Pump surfaces can tolerate rubbing friction for TBD msec without catastrophic failure). | X              | X       |
|   | 2. Cavitation or bubble/gas ingestion due to blocked or restricted inlet (contamination, or partial inlet valve failure, or failure in propellant acquisition system.) | 2. Reduced pump life, reduced thrust. HM directed shutdown due to pump overspeed. Restart is possible if overspeed did not reach design limit.  | X              | X       |

C-2

OTV DUAL PROPELLANT EXPANDER CYCLE ENGINE  
FAILURE MODES AND EFFECTS ANALYSIS

SYSTEM FAILURE  
FULL PARTIAL

C-3

| COMPONENT OR<br>SUBSYSTEM   | FAILURE MODE   | SYSTEM EFFECT   | FULL | PARTIAL   |
|---|--|---|------|-----------|
| Fuel Turbopump<br>Assembly  | 3. Turbine blade cracks, failure, or damage.   | 3. HM directed shutdown or reduced thrust operation.  | X    | X         |
|   | 1. Loss of LH <sub>2</sub> to hydrostatic bearings (contamination).  | 1. Bearings have redundant lubrication passages-partial loss may result in bearing wear, reduced pump life. Total loss will result in pump failure. |      | X         |
|   | 2. Cavitation or bubble/ingestion due to blocked or restricted inlet (contamination or partial inlet valve failure or failure in propellant acquisition system). | 2. Reduced pump life, reduced thrust. HM directed shutdown due to pump overspeed. Restart is possible if overspeed did not reach design limit.      | X    | X         |
| Oxidizer Boost Pump   | 3. Turbine blade cracks, failure, or damage.   | 3. HM directed shutdown or reduced thrust operations.   | X    | X         |
|   | 1. Pump fails to operate.  | 1. Requires tank head start - questionable reliability  |      | Uncertain |
|   | 2. Pump operates at reduced flow or pressure.  | 2. Engine MR, thrust, Isp are effected.   |      | X         |
|   | 3. 1 pump to operate or operates at reduced flow or pressure.  | 3. No effect - parallel redundant.  |      |           |
| <u>Concept A:</u><br>Individual boost pumps for each OTV engine.<br><u>Concept B:</u><br>2 ea. boost pumps feeding engine cluster inlet manifold. | 4. Both pumps fail to operate.   | 4. It successfully started on tank head pressure autogenous system will give normal operation.  |      | Uncertain |

OTV DUAL PROPELLANT EXPANDER CYCLE ENGINE  
FAILURE MODES AND EFFECTS ANALYSIS

C-4

| COMPONENT OR<br>SUBSYSTEM  | FAILURE MODE   | SYSTEM EFFECT  | SYSTEM FAILURE |         |
|--|--|--|----------------|---------|
|  |  |  | FULL           | PARTIAL |
| High Pressure Pump<br>Discharge Line(s)  | 1. Leaks or cracks at lines, seals, or joints.                                       | 1. Engine operates off MR. Lowr thrust and lsp   |                | X       |
| Hydrogen system only   | 2. Catastrophic structural failure or rupture.                                       | 2. Engine fails to opeate due to insufficient fuel.  | X              |         |
| Oxygen system only   | 3. Catastrophic structural failure.  | 3. Possible fire, low order explosion.   | X              |         |
| Igniter System<br>(Dual-redundant spark gap type elements. Inductive ignition coils, and electronics - may utilize hydrogen cooled ceramic element in lieu of igniter valves.) | 1. 1 unit fails to operate.  | 1. No effect - fully redundant system.   |                |         |
|  | 2. Both units fail to operate.   | 2. No engine ignition.   | X              |         |
|  | 3. Spark delay on startup.   | 3. Pressure spike and HM directed engine shutdown.   | X              |         |
| Oxygen Turbine Bypass Valve  | 1. Valves fail to open, close, or partially operate; or response time out of limits. | 1. Loss of throttling capability on affected engine. In multiple engine concept, other engines can gimbal and/or throttle to correct or null thrust vector. Health monitoring system can detect valve operational failure and adjust fuel throttling valve(s) to prevent catastrophic MR and resulting TCA damage. |                | X       |
|  | 2. Valve surfaces have rubbing contact or fretting.                                  | 2. Possible ignition depending on magnitude and duration of friction. HM directed engine shutdown.   | X              | TBD     |

OTV DUAL PROPELLANT EXPANDER CYCLE ENGINE  
FAILURE MODES AND EFFECTS ANALYSIS

| COMPONENT OR SUBSYSTEM  | FAILURE MODE   | SYSTEM EFFECT   | SYSTEM FAILURE |         |
|-------------------------|--|---|----------------|---------|
|                         |  |   | FULL           | PARTIAL |
| Hydrogen Turbine        | 3. Valve has internal leakage.   | 3. Magnitude of leakage may affect throttling capability and MR.  |                | X       |
|                         | 1. Valve fails to open, close, or partially operate; or response time out of limits. | 1. Loss of throttling capability on affected engine. In multiple engine concept, other engines can gimbal and/or throttle to correct or null thrust vector. Health monitoring system can detect valve operational failure and adjust oxidizer bypass valve to prevent catastrophic MR and resulting TCA damage. |                | X       |
|                         | 2. Valve has internal and/or external leakage.                                       | 2. Magnitude of leakage may affect throttling capability and MR, reduced Isp.   |                | X       |
| Thrust Chamber Assembly | 3. Failure of valve actuating coil control.  | 3. No effect. Redundant operating coils/ controls.  |                |         |
|                         | 1. Leaks or cracks on TCA inner walls.   | 1. System may operate off MR, reduced thrust and Isp.   |                | X       |
|                         | 2. Leaks or cracks on TCA outer walls.   | 2. System operates off MR, reduced thrust and Isp. Shutdown if leak detection system is installed.  | X              | X       |
|                         | 3. Major rupture or substantial foreign object damage.                               | 3. Engine shutdown by HM system.  | X              |         |
|                         | 4. Plugged cooling channels (contamination).   | 4. May reduce engine life, otherwise no effect on performance.  |                | X       |

OTV DUAL PROPELLANT EXPANDER CYCLE ENGINE  
FAILURE MODES AND EFFECTS ANALYSIS

| COMPONENT OR SUBSYSTEM                               | FAILURE MODE  | SYSTEM EFFECT  | SYSTEM FAILURE |         |
|--|---|--|----------------|---------|
|  |   |  | FULL           | PARTIAL |
| Baffle Plate   | 1. Burn through in outer wall.  | 1. May affect MR, Isp depending on magnitude.  |                | X       |
|  | 2. Plugged cooling channels (contamination).  | 2. May reduce baffle life - otherwise no effect on performance.  |                | X       |
| Injector   | 1. Plugged orifices (contamination).  | 1. May affect MR, Isp, thrust. Worst case (loss of fuel film cooling) may result in reduced engine life (thermal limits exceeded) or could cause TCA burn through. HM directed shutdown. | TBD            | X       |
| Engine Controller/<br>Health Monitoring<br>System    | 1. HM system erroneously predicts engine/system, component degradation or failure.                            | 1. Engine operates at lower thrust, Isp, or off MR. May be directed to shutdown.   | X              | X       |
|  | <u>Concept A:</u><br>2. Controller fails to adjust engine operation based on actual (non-catastrophic fault). | 2. Reduced engine life. May cause HM directed shutdown.  | X              | X       |
|  | <u>Concept B:</u><br>3. Partial failure of controller, or loss of instrumentation parameter.                  | 3. No effect - controller is redundant channel, can synthesize missing parameter or substitute nominal performance value.  |                |         |
|  | 4. Total failure of controller or instrumentation group.  | 4. Loss of engine operation. Propellant shut-off valves close.   | X              |         |
| Supervisory or limited control.                      |   |  |                |         |
| Full authority control (redundant multiple channel). |   |  |                |         |

C-16

OTV DUAL PROPELLANT EXPANDER CYCLE ENGINE  
FAILURE MODES AND EFFECTS ANALYSIS

| COMPONENT OR SUBSYSTEM     | FAILURE MODE   | SYSTEM EFFECT  | SYSTEM FAILURE |         |
|----------------------------|--|--|----------------|---------|
|                            |  |  | FULL           | PARTIAL |
| Instrumentation Transducer | 1. Transducer fails totally, calibration shifts, or transducer gives incorrect readings and/or transients. | 1. No effect - health monitoring system can synthesize parameter or substitute nominal performance value. Most transducers will be redundant dual element design; health monitoring system can "vote out" incorrect reading by rate-of-change transient computations (transducer drop outs or erratic transients) and by comparison with other combustion transducer parameters. | No effect      |         |
|                            |  |  |                |         |
| Gimbal System              | 1. Gimbal actuators fail to extend or retract, insufficient stroke, erratic operation or slew rate.        | 1. Unaffected engines (multiple engine scenario) can compensate by selective gimbaling or throttling to null or correct thrust vector.   |                | X       |
| Radiation Cooled Nozzle    | 1. Burn throughs, coating cracks, structural damage from handling, foreign objects/meteors.                | 1. Reduced thrust, Isp - otherwise no effect on engine system operation (throttling and/or gimbal system can compensate at system (multiple engine) level in worst case. Possible local heating of aerobrake.  |                | X       |
| Nozzle Retraction          | 1. Fails to retract.   | 1. May prevent aerobraking. Jettison if system has that capability.  | TBD            |         |
|                            | 2. Fails in intermediate position.   | 2. May prevent aerobraking. Jettison if system has that capability.  | TBD            |         |
| Nozzle Extension           | 1. Fails to extend.  | 1. Engine can be operated. No gimbal capability. Some local heating of aerobrake.  |                | X       |

C-7



|   |  |  |  |   |            |
|---|--|--|--|---|------------|
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| 16. Abstract<br>To insure controllability of the baseline design for a 7,500 pound thrust, 10:1 throttleable, dual expander cycle, Hydrogen-Oxygen, orbit transfer rocket engine, an Integrated Controls and Health Monitoring concept was developed. This included: (1) Dynamic engine simulations using a TUTESIM derived computer code, (2) Analysis of various control methods, (3) Failure Modes Analysis to identify critical sensors, (4) Survey of applicable sensors technology, and (5) Study of Health Monitoring philosophies. The engine design was found to be controllable over the full throttling range by using 13 valves, including an oxygen turbine bypass valve to control mixture ratio, and a hydrogen turbine bypass valve, used in conjunction with the oxygen bypass, to control thrust. Classic feedback control methods are proposed along with specific requirements for valves, sensors, and the controller. Expanding on the control system, a Health Monitoring system is proposed including suggested computing methods and the following recommended sensors: (1) Fiber optic and silicon bearing deflectometers, (2) Capacitive shaft displacement sensors, and (3) Hot spot thermocouple arrays. Further work is needed to refine and verify the dynamic simulations and control algorithms, to advance sensor capabilities, and to develop the Health Monitoring computational methods. |  |  |  |   |            |
| 17. Key Words (Suggested by Author(s))<br>Rocket Engine Controls, Health Monitoring,<br>Dual Expander Cycle, Orbit Transfer<br>Vehicle, Advanced Sensors  |  |  |  | 18. Distribution Statement<br>Unlimited                             |            |
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