Computer Code for Preliminary Sizing Analysis of Axial-Flow Turbines

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Summary

This report presents a computer program for the preliminary sizing analysis of axial-flow turbines. The computations are based on mean-diameter flow properties and a stage-average velocity diagram. Given as input to the program are power or pressure ratio, mass flow rate, inlet temperature and pressure, rotative speed, inlet and exit diameters (either hub, mean, or tip), exit radius ratio or stator exit angle, turbine loss coefficient, and gas properties. Computations are then performed for any specified number of stages and for any of three types of velocity diagrams (symmetrical, zero exit swirl, or impulse) or for any specified stage swirl split. Exit turning vanes can be included in the design. The program output includes inlet and exit annulus dimensions, exit temperature and pressure, total and static efficiencies, flow angles, and last-stage absolute and relative Mach numbers.

The analysis and code presented herein are modifications of those in reference 1. New features added to improve modeling rigor and extend code applicability include a generalized velocity diagram, a more flexible meanline path, a reheat model, a radial component of velocity, and a computation of free-vortex hub and tip velocity diagrams. Also, a loss-coefficient calibration was performed to provide recommended values for current technology airbreathing engine turbines.

The analysis method and a complete description of input and output are presented in this report. Sample cases are included to illustrate the use of the program. Because reference 1 is no longer readily available, this report describes the entire analysis, not only the modifications.

Symbols

- $A$ stage loss parameter
- $A_{an}$ annulus area, $m^2$; $ft^2$
- $C$ blade loss parameter
- $C_A$ dimensional constant, $2\pi$ rad/rev; 60 sec/min
- $C_B$ dimensional constant, 1; 550 ft · lb/sec · hp
- $c_x$ axial chord, $m$; $ft$
- $c_p$ heat capacity, joules/kg · K; Btu/lb · °R
- $D$ diameter, $m$; $ft$
- $E$ squared ratio of stage-exit axial velocity to stage-average axial velocity
- $F$ blade loss weighting factor
- $g$ dimensional constant, 1; 32.2 ft · lbm/sec$^2$ · lbf
- $\Delta h$ specific work, J/kg; Btu/lb
- $i$ stage number, $i = 1, 2, \ldots, n$
- $J$ dimensional constant, 1; 778 ft · lb/Btu
- $k$ coefficients for axial chord correlation
- $M$ Mach number
Method of Analysis

The method is based upon an analysis of the flow at the turbine mean diameter. Radial gradients of the flow properties are not included in the flow and efficiency analyses. However, free-vortex hub and tip velocity-diagram parameters are computed in order to indicate the possible severity of flow conditions at the endwalls. The specific heat ratio is assumed constant throughout the turbine. For any given turbine, all stages, except the first, are specified to have the same shape velocity diagram. The first stage differs only in that the inlet flow can be at any specified flow angle. The velocity diagram shape depends upon the stage work factor value and the specified velocity diagram. Three specific types of velocity diagram (symmetrical, zero exit swirl, and impulse) or a diagram with a specified swirl split (i.e., distribution of swirl velocity between rotor inlet and rotor exit) can be used. Figure 1 illustrates the velocity diagram symbolism and presents the three specific types of velocity diagram for three values of stage work factor. The fundamentals of velocity diagrams and of the flow and loss modeling used herein can be found in reference 3.

Various input options dictate the exact nature of the calculation procedure. There is, however, one basic procedure that is direct and without iteration. This basic procedure will be presented, and then the alternative procedures required for the various input options will be discussed.

Basic Calculation Procedure

The required inputs for the basic procedure are shaft power $P$, mass flow rate $w$, inlet total temperature $T_{in}$, inlet total pressure $p_{in}$, rotative speed $N$, inlet mean diameter $D_{m,in}$, exit mean diameter $D_{m,ex}$, stator exit angle $\alpha_1$, gas constant $R$, specific heat ratio $\gamma$, viscosity $\mu$, loss coefficient $K$, and the squared ratio of stage-exit to stage-average (i.e., $\frac{V_{me,0} + V_{me,1} + V_{me,2}}{3}$) meridional velocities $E$. Also specified for each calculation are the number of stages $n$ and the type of velocity diagram. For a multistage turbine, the input variable specified as inlet diameter is used to calculate first rotor blade speed and annulus dimensions at the first rotor exit. Therefore, it is truly an inlet diameter only if the hub and tip diameters
are assumed constant across the first stage. For a one-stage turbine, the specified exit diameter is used for the calculations, and the inlet diameter is of no significance.

The first- and last-stage blade speeds are 

\[ U_a = \frac{\pi ND_m, in}{C_A} \quad (1) \]

\[ U_n = \frac{\pi ND_m, ex}{C_A} \quad (2) \]

For more than two stages, there are three options for stage blade-speed variation (i.e., mean flowpath):

1. Linear variation between first and last stages

\[ U_i = \frac{U_n - U_a}{n - 1} (i - 1) + U_a \quad (3a) \]

2. Constant from first stage to \( j \)th stage (\( i = 1 \) to \( j \))
\[
\eta = \frac{\Delta h'}{\sum_{i=1}^{n} \frac{\Delta h_{id,i}'}{\sum_{i=1}^{n} \left( \frac{T_{id,i}}{T'} \right)^2 \Delta h_{id,i}' / \Delta h'}
\]

or alternately expressed as

\[
\eta = \frac{1}{\sum_{i=1}^{n} \left( \frac{T_{id,i}}{T'} \right)^2 \frac{\Delta h_{id,i}'}{\sum_{i=1}^{n} \frac{\Delta h_{id,i}'}{\Delta h'}}
\]

Combining equations (6) and (7) yields

\[
\frac{\Delta h_{id,i}'}{\sum_{i=1}^{n} \frac{\Delta h_{id,i}'}{\Delta h'}} = \frac{U_i^2}{\sum_{i=1}^{n} \frac{U_i^2}{\Delta h'}}
\]

and, by definition

\[
\psi = \frac{gJ \Delta h'}{\sum_{i=1}^{n} \frac{U_i^2}{\Delta h'}}
\]

The value of \( \psi \) is the primary factor determining turbine efficiency.

The method used for computing turbine efficiency is basically similar to that presented in reference 2, but has the following additional features: (1) the turbines considered in this report are not restricted to a constant mean-section diameter, (2) exit vanes to provide axial flow leaving the turbine can be included in the design, (3) the velocity diagrams also can be specified as symmetrical or by an input swirl split, and (4) the effect of turbine reheat is included.

Referring to figure 2, turbine overall total efficiency is defined as
Substituting equations (10) and (11) into equation (9b) yields

\[
\eta' = \frac{1}{\sum \left( \frac{T_{i+1}^*}{T_i^*} \right) \left( \frac{U_i^2}{\eta_i^*} \right) \left( \sum U_i^2 \right)}
\]

The terms \( T_i^* \) and \( T_{i+1}^* \) can be written as

\[
T_i^* = T_{i-1}^* - \frac{\Delta h_i^*}{c_p} = T_{i-1}^* - \frac{U_i^2}{g J c_p} \psi
\]

\[
T_{i+1}^* = T_{i+1}^* - \frac{\Delta h_{i+1}}{c_p} = T_{i+1}^* - \left( \frac{T_{i+1}^*}{T_i^*} \right) \left( \frac{1}{\eta_i^*} \right) \frac{U_i^2}{g J c_p} \psi
\]

where

\[
c_p = \frac{\gamma R}{\gamma - 1} \frac{1}{J}
\]

Since both \( T_i^* \) and \( T_{i+1}^* \) are equal to \( T_{in}^* \) at the turbine inlet (i.e., at \( i = 1 \)), equations (13) and (14) can be evaluated by recursion once the \( \eta_i^* \) are known.

The stage efficiency computation method is explained fully in reference 2, and only the key equations are presented here. Stage total efficiency can be expressed as

\[
\eta' = \frac{1}{1 + A \psi}
\]

where the stage loss parameter \( A \) is

\[
A = \frac{K \Re^{-0.2}}{\cot \alpha_1} (F_{\text{st}} C_{\text{st}} + F_{\text{ro}} C_{\text{ro}} + C_{\text{ev}})
\]

The constant of proportionality \( K \), called the turbine loss coefficient in this report, must be determined empirically, and recommended values based on experimental efficiencies are presented later in this report. The Reynolds number used in this calculation is defined as

\[
\Re = \frac{2w}{\mu D_{m,ja}}
\]

Some of the terms within the parentheses in equation (17) are the same for all cases, while others depend on stage location, velocity diagram type, and use of exit vanes. The rotor weighting factor \( F_{\text{ro}} \) and rotor loss parameter \( C_{\text{ro}} \) are the same for all cases:

\[
F_{\text{ro}} = 2 \quad (19)
\]

\[
C_{\text{ro}} = 2 \cot^2 \alpha_1 \left( \frac{V_{u,1}}{\Delta V_u} \right)^2 + \left( \frac{V_{u,2}}{\Delta V_u} - \frac{1}{\psi} \right)^2
\]

(20)

For last stages of turbines having exit vanes,

\[
C_{\text{ev}} = 2 \cot^2 \alpha_1 \left( \frac{V_{u,1}}{\Delta V_u} \right)^2 + \left( \frac{V_{u,2}}{\Delta V_u} \right)^2
\]

(21)

For all other stages

\[
C_{\text{ev}} = 0
\]

(22)

For first-stage stators having a specified inlet angle \( \alpha_0 \), the stator loss parameter is expressed

\[
C_{\text{st}} = \left[ 1 + \cot^2 \alpha_1 \left( 2 + \tan^2 \alpha_0 \right) \right] \left( \frac{V_{u,1}}{\Delta V_u} \right)^2
\]

(23)

For all other stators

\[
C_{\text{st}} = (1 + 2 \cot^2 \alpha_1) \left( \frac{V_{u,1}}{\Delta V_u} \right)^2 + \left( \frac{V_{u,2}}{\Delta V_u} \right)^2
\]

(24)

The stator weighting factor \( F_{\text{st}} \) also depends on whether or not the stator is a first-stage stator. For a first-stage stator having a specified inlet angle \( \alpha_0 \),

\[
F_{\text{st}} = \frac{1 - 3 \tan \alpha_0}{\tan \alpha_1}
\]

(25)

For all other stators

\[
F_{\text{st}} = \frac{1 - 3(V_{u,2}/\Delta V_u)}{V_{u,1}/\Delta V_u}
\]

(26)
The rotor inlet and exit swirl parameters $V_{u,1}/\Delta V_u$ and $V_{u,2}/\Delta V_u$ of equations (19) to (26) depend on the specific type of velocity diagram or are input for the general diagram. The following table presents the relations for evaluating the swirl parameters:

<table>
<thead>
<tr>
<th>Velocity</th>
<th>Inlet swirl parameter, $V_{u,1}/\Delta V_u$</th>
<th>Exit swirl parameter, $V_{u,2}/\Delta V_u$</th>
</tr>
</thead>
<tbody>
<tr>
<td>Symmetrical</td>
<td>$\frac{1 + \psi}{2\psi}$</td>
<td>$\frac{1 - \psi}{2\psi}$</td>
</tr>
<tr>
<td>Zero exit swirl</td>
<td>1</td>
<td>0</td>
</tr>
<tr>
<td>Impulse</td>
<td>$\frac{2 + \psi}{2\psi}$</td>
<td>$\frac{2 - \psi}{2\psi}$</td>
</tr>
<tr>
<td>General</td>
<td>Input</td>
<td>$\frac{V_{u,1}}{\Delta V_u} - 1$</td>
</tr>
</tbody>
</table>

With turbine total efficiency obtained from the foregoing equations, the turbine exit velocities and state conditions are computed as follows:

$$\Delta V_{u,n} = \psi U_n$$  \hspace{1cm} (27)

$$V_{u,1,n} = \frac{V_{u,1}}{\Delta V_u} \Delta V_{u,n}$$  \hspace{1cm} (28)

$$V_{u,2,n} = \frac{V_{u,2}}{\Delta V_u} \Delta V_{u,n}$$  \hspace{1cm} (29)

$$V_{me,n} = V_{u,1,n} \cot \alpha_1$$  \hspace{1cm} (30)

$$V_{me,ex} = \sqrt{E} V_{me,n}$$  \hspace{1cm} (31)

$$V_{u,ex} = \begin{cases} V_{u,2,n} & \text{no exit vanes} \\ 0 & \text{exit vanes} \end{cases}$$  \hspace{1cm} (32)

$$V_{ex} = \sqrt{V_{me,ex}^2 + V_{n,ex}^2}$$  \hspace{1cm} (33)

$$p_{ex} = p_{in} \left(1 - \frac{\Delta h'}{\sqrt{c_p T_{in} \gamma}}\right)^\gamma (34)$$

$$T_{ex} = T_{in} \frac{\Delta h'}{\varepsilon_p}$$  \hspace{1cm} (35)

$$T_{ex} = T_{ex}' \frac{V_{ex}^2}{2gJc_p}$$  \hspace{1cm} (36)

$$p_{ex} = p_{ex} \left(\frac{T_{ex}}{T_{ex}'}\right)^\gamma (37)$$

In order that the turbine total and static efficiencies be consistent with the computed exit velocity, the static efficiency is computed as:

$$\eta = \frac{\Delta h'}{c_p T_{in} \left[1 - \left(\frac{p_{ex}}{p_{in}}\right)^{\gamma (\gamma - 1)}\right]}$$  \hspace{1cm} (38)

In order to calculate continuity for a flowpath having significant inclination, it is necessary to determine a flowpath slope. Consistent with the preliminary nature of this analysis, a correlation relating axial chord to mean diameter was used to determine axial length. The correlation, based on the geometry of existing turbines, is of the form

$$c_x = k_1D_m + k_2D_m^2 + k_3 + k_4D_m^{-1}$$  \hspace{1cm} (39)

and was established over a range of mean diameters from 0.10 to 1.25 m (0.33 to 4.17 ft). Because of the scatter in the available data, correlations were determined for low, mean, and high aspect ratios. The correlation coefficients are presented in table I. The three correlation curves for ratio of axial chord to mean diameter are plotted in figure 3. Using one-third of axial chord length for spacing between blade rows, the slope for a linear meanline between stage 1 exit and stage $n$ exit is

$$\tan \theta = \frac{0.5(D_{m,ex} - D_{m,in})}{n} = \frac{3(D_{m,ex} - D_{m,in})}{2}$$  \hspace{1cm} (40)

For two-segment meanlines, the lower or upper limit for the summation is changed as appropriate for the linear section, and the slope for the constant section is set equal to zero.
Exit annulus area, radius ratio, and hub and tip diameters are obtained as follows:

\[
\rho_{ex} = \frac{\rho_{ex}}{RT_{ex}} \tag{41}
\]

\[
V_{x,ex} = V_{me,ex} \cos \theta_{in} \tag{42}
\]

\[
A_{ap,ex} = \frac{w}{\rho_{ex} V_{x,ex}} \tag{43}
\]

\[
\left( \frac{r_2}{r_1} \right)_{ex} = \frac{1 - \frac{A_{ap,ex}}{\pi D_m^{2,ex}}}{1 + \frac{A_{ap,ex}}{\pi D_m^{2,ex}}} \tag{44}
\]

\[
D_{x,ex} = \frac{2D_{m,ex}}{1 + \left( \frac{r_2}{r_1} \right)_{ex}} \tag{45}
\]

\[
D_{h,ex} = D_{x,ex} \left( \frac{r_2}{r_1} \right)_{ex} \tag{46}
\]

Absolute and relative flow angles, which are the same for each stage, are computed from the last-stage velocities:

\[
\alpha_2 = \tan^{-1} \frac{V_{u,2,n}}{V_{me,n}} \tag{47}
\]

\[
W_{u,1,n} = V_{u,1,n} - U_n \tag{48}
\]

\[
W_{u,2,n} = V_{u,2,n} - U_n \tag{49}
\]

\[
\beta_1 = \tan^{-1} \frac{W_{u,1,n}}{V_{me,n}} \tag{50}
\]

\[
\beta_2 = \tan^{-1} \frac{W_{u,2,n}}{V_{me,n}} \tag{51}
\]

\[
V_{2,n} = \sqrt{V_{u,2,n}^2 + V_{me,ex}^2} \tag{52}
\]

\[
W_{2,n} = \sqrt{W_{u,2,n}^2 + V_{me,ex}^2} \tag{53}
\]

\[
W_{1,n} = \sqrt{W_{u,1,n}^2 + V_{me,n}^2} \tag{54}
\]

\[
V_{1,n} = \sqrt{V_{u,1,n}^2 + V_{me,n}^2} \tag{55}
\]

Figure 3.—Axial chord correlation.
Efficiency and continuity computations are based solely on the meanline flow parameters. Although radial variations of velocity in a turbine need not be of the free-vortex type, the free-vortex hub and tip velocity-diagram parameters can give a qualitative indication of the possible severity of end wall flow conditions. With blade speed directly proportional to diameter

\[ U \propto D \]  

and applying the free-vortex conditions for radial variations of the velocity components

\[ V_u \propto \frac{1}{D} \]  

and

\[ V_{me} = \text{Constant} \]

the hub and tip velocity diagrams are determined from equations (47) to (61).

In order to establish the flow annulus geometry near the turbine inlet, a flow analysis is made at the first-stage exit as follows:

\[ \Delta h_u = \frac{U_u^2}{8g} \psi \]  

\[ T_2,a = T_{in} - \frac{\Delta h_u}{c_p} \]  

\[ p_{2,a} = P_{in} \left( 1 - \frac{\Delta h_u}{c_p T_{in} \psi} \right)^{\gamma/(\gamma - 1)} \]  

\[ \Delta V_{u,a} = \psi U_u \]

When a constant annulus is assumed for the first stage, the first-stage exit dimensions become the inlet dimensions for that turbine.

**Alternative Calculation Procedure**

The basic calculation procedure described in the previous section requires as inputs the inlet and exit mean diameters, stator exit angle, and shaft power. Alternatively, the hub or tip diameters could be specified as input, and the mean diameters computed; the exit radius ratio could be specified as input, and the stator exit angle computed; and/or the turbine pressure ratio could be specified as input, and the shaft power computed. These alternative input options require iterative calculation procedures such as described in this section.
With hub or tip diameters rather than mean diameters specified at the inlet and exit, it is necessary to assume initial values for the inlet and exit radius ratios. Initial values for inlet and exit mean diameters are then obtained as

\[ D_m = \frac{\left(1 + \frac{r_h}{r_t}\right)D_h}{2} \]  

(80)

when hub diameter is input and

\[ D_m = \frac{\left(1 + \frac{r_t}{r_h}\right)D_t}{2} \]  

(81)

when tip diameter is input. The computation then proceeds from equation (1) to equation (44), and the computed exit radius ratio is compared with the assumed value. If they are not the same (within a given tolerance), then the computed value of exit radius ratio is used to calculate a new value for exit mean diameter (from eq. (80) or (81)), and the computation procedure is repeated until convergence is obtained. Then, computation proceeds through equation (77), and the computed inlet radius ratio is compared with the assumed value. If they are not within tolerance, the computed value of inlet mean diameter is used, and the computation procedure is repeated from equation (1). This entire procedure is repeated until both inlet and exit radius ratios converge in the same calculation pass. When hub or tip diameter rather than mean diameter is used as input, a two-segment meanline cannot be specified.

With exit radius ratio rather than stator exit angle specified, a value of stator exit angle is assumed for the evaluation of equation (17). The computation proceeds through equation (41). Equation (44) is then used to compute the exit annulus area from the input value of radius ratio, and the exit axial velocity is then obtained from equation (43). The density used in equation (43), however, is not consistent with the exit area, and equations (43), (42), (33), (36), and (41) must be iterated until convergence is obtained. Then, the stator exit angle is computed as

\[ \alpha_i = \tan^{-1}\frac{V_{u,i,n}}{V_{me,n}} \]  

(82)

and compared with the assumed value. If they are not within tolerance, the computed value of stator exit angle is used for the evaluation of equation (17), and the computation procedure just given is repeated until stator exit angle converges. The remainder of the computation is then completed.

With the turbine inlet-total- to exit-static-pressure ratio rather than shaft power specified as input, an initial value of turbine static efficiency is assumed. Turbine work is then computed from

\[ \Delta h' = \eta_c \frac{p_{ex}}{p_{in}} \left[ 1 - \left(\frac{p_{ex}}{p_{in}}\right)^{(\gamma-1)/\gamma} \right] \]  

(83)

instead of from equation (4). The computation then proceeds through equation (38) to compute a static efficiency. If the computed and the assumed values are not within tolerance, a new value of static efficiency is assumed, and the computation is repeated until convergence. The remainder of the computation is then completed.

### Loss-Coefficient Evaluation

Ten turbines that were designed for airbreathing engine applications and performance tested in component facilities were selected to serve as the basis for the loss coefficient evaluation. These turbines represent the output of several different design systems and cover a wide range of design characteristics. Six of the turbines were designed and tested by NASA, two were designed and tested by GE Aircraft Engines Co., and two were designed by Pratt & Whitney, one of which was tested by Pratt & Whitney and the other by NASA. The characteristics of these turbines are presented in table II. The number of stages varies from 1 to 5, stage work factors \(\psi\) from 1.2 to 4.7, stator angles from 51° to 80°, exit mean diameters, except for one, from 47 to 69 in. (18 to 27 in.), and radius ratios from 0.50 to 0.86. All the listed total efficiencies except for two were as measured. In those two cases, corrections were made to account for the effects of cooling air (ref. 4) and a large discrepancy in flow rate from design intent (ref. 5).

These turbines were modeled as closely as possible for the subject computer program, and design performance was computed over a range of loss coefficients. The variation of efficiency with loss coefficient was a direct proportion between the quantity \(1 - \eta\) and loss coefficient, \(K\). Calculated efficiencies are plotted against measured efficiencies in figure 4 for a constant loss coefficient of 0.3. As seen, for all cases except one, the calculated efficiency was within one point of the measured efficiency. In view of the simplistic modeling used for this computer program, this agreement is admittedly fortuitous. It does illustrate, however, that this program is capable of predicting reasonable efficiencies over a wide range of airbreathing engine turbine conditions. A loss coefficient value of 0.3 is recommended for use in the absence of additional information.
TABLE II.—TURBINES USED FOR EVALUATION

<table>
<thead>
<tr>
<th>Number of stages</th>
<th>Average stage work factor, ( \psi )</th>
<th>Average stator angle, deg</th>
<th>Exit mean diameter</th>
<th>Exit radius ratio</th>
<th>Measured efficiency</th>
<th>Reference</th>
</tr>
</thead>
<tbody>
<tr>
<td>1</td>
<td>1.7</td>
<td>67.0</td>
<td>66.0 26.0</td>
<td>0.73</td>
<td>0.923</td>
<td>6</td>
</tr>
<tr>
<td>2</td>
<td>1.7</td>
<td>64.5</td>
<td>66.0 26.0</td>
<td>0.63</td>
<td>0.932</td>
<td>7</td>
</tr>
<tr>
<td>1</td>
<td>1.2</td>
<td>63.4</td>
<td>20.3 8.0</td>
<td>0.68</td>
<td>0.917</td>
<td>8</td>
</tr>
<tr>
<td>1</td>
<td>1.9</td>
<td>73.1</td>
<td>47.0 18.5</td>
<td>0.85</td>
<td>0.886</td>
<td>9</td>
</tr>
<tr>
<td>3.5</td>
<td>4.0</td>
<td>51.4</td>
<td>49.8 19.6</td>
<td>0.50</td>
<td>0.880</td>
<td>5</td>
</tr>
<tr>
<td>4.5</td>
<td>4.7</td>
<td>66.6</td>
<td>49.8 19.6</td>
<td>0.57</td>
<td>0.855</td>
<td>10</td>
</tr>
<tr>
<td>1</td>
<td>1.7</td>
<td>75.0</td>
<td>46.7 18.4</td>
<td>0.86</td>
<td>0.895</td>
<td>11</td>
</tr>
<tr>
<td>1</td>
<td>1.6</td>
<td>79.6</td>
<td>56.4 22.2</td>
<td>0.85</td>
<td>0.901</td>
<td>12</td>
</tr>
<tr>
<td>2</td>
<td>1.3</td>
<td>71.6</td>
<td>69.3 27.3</td>
<td>0.82</td>
<td>0.898</td>
<td>13</td>
</tr>
<tr>
<td>5</td>
<td>2.6</td>
<td>61.0</td>
<td>63.5 25.0</td>
<td>0.92</td>
<td>0.920</td>
<td>13</td>
</tr>
</tbody>
</table>

Description of Input and Output

This section presents a detailed description of the program input, normal output, and error messages. Included in the input and output sections are several example cases illustrating the use of the program and the various options.

Input

The program input, a sample of which is presented in table III, consists of a title record and the required physical data and option indicators in NAMELIST form. The title, which is printed as a heading on the output listing, can contain up to 77 characters located anywhere in columns 2 to 78 on the title record. A title, even if it is left blank, must be the first record of the input data. Additional titles can be used to identify different cases being run in the same data file. This is done by placing a title in front of the data for the particular case and using the option indicator ITIT as subsequently described.

The physical data and option indicators are input in data records having the NAMELIST name INPUT. The variables and indicators that compose INPUT and the proper units are as follows. These must be input for all cases except where otherwise indicated. Either the SI units or the U.S. customary units shown below may be used.

- **PTIN**: inlet total pressure, \( \text{N/cm}^2; \text{lb/in.}^2 \)
- **TTIN**: inlet total temperature, \( K; ^{\circ}\text{R} \)
- **MU**: gas viscosity, \( \text{N} \cdot \text{sec/m}^2; \text{lb/sec} \cdot \text{ft} \)
- **R**: gas constant, \( \text{J/kg} \cdot \text{K}; \text{ft} \cdot \text{lbf/lbm} \cdot ^{\circ}\text{R} \)
- **GAM**: specific heat ratio
- **DIN**: inlet diameter—hub or mean or tip value as specified by the indicator IDIAM, cm; in.
- **DEX**: exit diameter—hub or mean or tip value as specified by the indicator IDIAM, cm; in.
- **RREX**: exit radius ratio; RREX may be omitted in the case where both IDIAM = 2 and IALPH = 0; RREX is used as first trial value when IALPH = 0 and IDIAM = 1 or 3
- **RPM**: rotative speed, rad/sec; rpm
- **POW**: shaft power—omit when IPR = 1, k\( \text{W}; \text{hp} \)
- **W**: mass flow rate, kg/sec; lb/sec

These are sample cases for this program. U.S. Customary unit are used.

In the sample input, the following options are used:

- **PTIN**: inlet total pressure, N/cm²; lb/in.²
- **TTIN**: inlet total temperature, K; °R
- **MU**: gas viscosity, N·sec/m²; lb/sec·ft
- **R**: gas constant, J/kg·K; ft·lbf/ibm·°R
- **GAM**: specific heat ratio
- **DIN**: inlet diameter—hub or mean or tip value as specified by the indicator IDIAM, cm; in.
- **DEX**: exit diameter—hub or mean or tip value as specified by the indicator IDIAM, cm; in.
- **RREX**: exit radius ratio; RREX may be omitted in the case where both IDIAM = 2 and IALPH = 0; RREX is used as first trial value when IALPH = 0 and IDIAM = 1 or 3
- **RPM**: rotative speed, rad/sec; rpm
- **POW**: shaft power—omit when IPR = 1, kW; hp
- **W**: mass flow rate, kg/sec; lb/sec

![Figure 4.—Comparison of measured and calculated total efficiencies. Loss coefficient, 0.3.](image-url)
ALPHA  stator exit angle from axial direction; ALPHA is used as first trial value when IALPH = 1, deg
ALPHA0  turbine inlet flow angle; required only when KALPH0 = 2, deg
VU1DVU ratio of rotor inlet swirl to total change in swirl; input only when IVD = 5
KLOSS  turbine loss coefficient; a value of 0.3 is recommended in the absence of additional information
NMIN  minimum number of stages for which the calculations are performed
NMAX  maximum number of stages for which the calculations are performed; results are obtained for all stage numbers between NMIN and NMAX
NMID  stage number at which meanline changes slope; may be omitted when IMID = 0
E  squared ratio of stage-exit to stage-average meridional velocities
PRTS  turbine inlet-total to exit-static-pressure ratio; omit when IPR = 0
IALPH indicates whether stator exit angle or turbine exit radius ratio is specified:
IALPH = 0—turbine is designed for specified ALPHA
IALPH = 1—turbine is designed for specified RREX
IDIAM indicates whether input diameters are hub, mean, or tip values:
IDIAM = 1—input diameters are hub values
IDIAM = 2—input diameters are mean values
IDIAM = 3—input diameters are tip values
IVD  indicates type of velocity diagram used:
IVD = 1—symmetrical diagrams
IVD = 2—zero exit swirl diagrams
IVD = 3—impulse diagrams
IVD = 4—zero exit swirl diagrams if $\psi \leq 2.0$ and impulse diagrams if $\psi \geq 2.0$
IVD = 5—ratio of rotor inlet swirl to total change in swirl is input as VU1DVU
ITIT indicates use of titles in addition to that required as first line of data package:
ITIT = 1-title line precedes next data set; must be input for each additional title because ITIT is automatically restored to zero after each title is read
IEV  indicates use of exit vanes:
IEV = 0—no exit vanes
IEV = 1—exit vanes are used to turn turbine exit flow to axial direction
IPR  indicates whether shaft power or pressure ratio is specified:
IPR = 0—shaft power is input
IPR = 1—turbine inlet-total- to exit-static-pressure ratio is input
IU  indicates type of units used for input and output:
IU = 1—SI units
IU = 2—U.S. customary units
KALPH0 indicates turbine-inlet flow angle option:
KALPH0 = 0—turbine-inlet flow is axial (default)
KALPH0 = 1—turbine-inlet flow angle equals stage-exit flow angle
KALPH0 = 2—turbine-inlet flow angle is input as ALPHA0
IAR indicates blading aspect ratio:
IAR = 1—high aspect-ratio blading
IAR = 2—mid aspect-ratio blading (default)
IAR = 3—low aspect-ratio blading
IMID indicates meanline shape:
IMID = 0—meanline linear from stage 1 to stage N (default)
IMID = 1—meanline constant from stage 1 to stage NMID and then linear to stage N
IMID = 2—meanline linear from stage 1 to stage NMID and then constant to stage N

The first line of the input file shown in table III is the mandatory title card, which can contain any desired message. The next three lines are the first data set, which contains all required inputs. This first case represents computation in accordance with the basic calculation procedure described previously. Data input for subsequent cases need only include those values that differ from previous case data. The fifth line is the second data set and represents the option where hub diameter is input. Also, the second case data specify that a title, which is the sixth data line, precede the third case data. Cards 7 to 10 represent four additional cases illustrating the use of different input options. The output corresponding to this sample input is described in the following section.

Output

The program output consists of title headings, the input variables, and computed results. This section presents normal output. Error message output is described in the next section.

Table IV presents the output that corresponds to the sample input shown in table III. The top line of output is a program identification title that is automatically printed. The second line is the title card message. The next four lines are the input variables and their associated values for the first data set. The input variable names are spelled out. The units for the input variable values are as described in the "Input" section. The zeros
printed under EXIT RADIUS RATIO and T-S PRESSURE RATIO indicate that these are computed for this case and not specified by the input. The fact that the input diameters are mean diameters is indicated by the MN in the variable name. Hub and tip diameters would be indicated by HB and TP, respectively. These four lines of output are printed for each new data set.

The next group of eight lines is the computation results satisfying the input requirements. The output parameters are spelled out and are self-explanatory. These temperatures, pressures, velocities, and angles are meanline values. On the first line are the number of stages, the stage work factor, the specific work is printed for the case where pressure ratio is specified as input.

Error Messages

The program contains seven output messages. Five of these indicate the nonexistence of a meanline solution satisfying the input requirements. The output parameters are spelled out and are self-explanatory. These temperatures, pressures, velocities, and angles are meanline values. On the first line are the number of stages, the stage work factor, the specific work is printed for the case where pressure ratio is specified as input.

After the computations for each input case are completed, the input data for the next case are printed. The second case presented here is that where the specified diameters are hub values. The third input case in Table IV is preceded by an additional title line. This causes the next output to begin at the top of a new page with the program identification and the title message.

The third case is for a specified exit radius ratio rather than for a specified stator exit angle. This is indicated by a zero appearing under STATOR EXIT ANGLE in the row of input variables. Exit vanes are included in this case, as indicated by WITH EXIT VANES printed after number of stages on the first line of result output. The fourth and fifth cases, as indicated by the top line of the fourth column of result output, are for zero exit swirl and impulse diagrams, respectively. Turbine pressure ratio rather than shaft power is specified in the sixth case. For this case, the input pressure ratio was chosen to be the same as the computed value obtained for the second case. As should be expected, the program converges to identical solutions for both cases. An extra line of output consisting of shaft power and specific work is printed for the case where pressure ratio is specified as input.

### TABLE IV.—SAMPLE OUTPUT

#### TURBINE VELOCITY DIAGRAM ANALYSIS

<table>
<thead>
<tr>
<th>STAGE</th>
<th>SHAFT MASS</th>
<th>INLET</th>
<th>INLET ROTATIVE</th>
<th>INLET EXIT RADIUS</th>
<th>STATOR GAS</th>
<th>CAPAC VISCOSITY</th>
<th>TURBINE AXIAL RADIUS</th>
<th>HEX GAS</th>
<th>TURBINE AXIAL T-S PRESS</th>
<th>COEF</th>
<th>RATIO RATIO</th>
</tr>
</thead>
<tbody>
<tr>
<td></td>
<td>0</td>
<td>53.50</td>
<td>2660.00</td>
<td>115.10</td>
<td>11400.00</td>
<td>65.00</td>
<td>53.37</td>
<td>1.300</td>
<td>0.376E-04</td>
<td>0.350</td>
<td>1.200</td>
</tr>
<tr>
<td>STAGES* 2</td>
<td>EXIT TIP DIAMETER = 27.18</td>
<td>EXIT HUB DIAMETER = 20.00</td>
<td>EXIT RADIUS RATIO = 0.7658</td>
<td>INLET TIP DIAMETER = 24.16</td>
<td>INLET RADIUS RATIO=0.8233</td>
<td>LAST STG M1 ABS = 0.8300</td>
<td>STAGE WORK FACTOR = 1.65</td>
<td>REYNOLDS NO.=0.1552E+07</td>
<td>DIAGRAMS ARE SYMMETRICAL</td>
<td></td>
<td></td>
</tr>
<tr>
<td>HUB: LAST STG M1 ABS = 0.0915</td>
<td>TIP: LAST STG M1 ABS = 0.7610</td>
<td>STATOR EXIT ANGLE = 67.98</td>
<td>LAST STG M1 REL = 0.4362</td>
<td>LAST STG M2 REL = 0.8684</td>
<td>LAST STG M2 ABS = 0.8764</td>
<td>LAST STG M2 M1 REL = 0.6551</td>
<td>LAST STG M2 M2 REL = 0.6651</td>
<td>LAST STG M2 M3 REL = 0.6651</td>
<td>LAST STG M2 M4 REL = 0.6651</td>
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<td></td>
</tr>
<tr>
<td>STAGE 1-2 MEANLINE SLOPE = 16.85 DEG BASED ON MID ASPECT-RATIO BLADING</td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
<td></td>
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<table>
<thead>
<tr>
<th>STAGE</th>
<th>SHAFT MASS</th>
<th>INLET</th>
<th>INLET ROTATIVE</th>
<th>INLET EXIT RADIUS</th>
<th>STATOR GAS</th>
<th>CAPAC VISCOSITY</th>
<th>TURBINE AXIAL RADIUS</th>
<th>HEX GAS</th>
<th>TURBINE AXIAL T-S PRESS</th>
<th>COEF</th>
<th>RATIO RATIO</th>
</tr>
</thead>
<tbody>
<tr>
<td></td>
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<td>2660.00</td>
<td>115.10</td>
<td>11400.00</td>
<td>65.00</td>
<td>53.37</td>
<td>1.300</td>
<td>0.376E-04</td>
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<td>1.200</td>
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<tr>
<td>STAGES* 2</td>
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<td>EXIT HUB DIAMETER = 20.00</td>
<td>EXIT RADIUS RATIO = 0.7658</td>
<td>INLET TIP DIAMETER = 24.16</td>
<td>INLET RADIUS RATIO=0.8233</td>
<td>LAST STG M1 ABS = 0.8300</td>
<td>STAGE WORK FACTOR = 1.65</td>
<td>REYNOLDS NO.=0.1552E+07</td>
<td>DIAGRAMS ARE SYMMETRICAL</td>
<td></td>
<td></td>
</tr>
<tr>
<td>HUB: LAST STG M1 ABS = 0.0915</td>
<td>TIP: LAST STG M1 ABS = 0.7610</td>
<td>STATOR EXIT ANGLE = 67.98</td>
<td>LAST STG M1 REL = 0.4362</td>
<td>LAST STG M2 REL = 0.8684</td>
<td>LAST STG M2 ABS = 0.8764</td>
<td>LAST STG M2 M1 REL = 0.6551</td>
<td>LAST STG M2 M2 REL = 0.6651</td>
<td>LAST STG M2 M3 REL = 0.6651</td>
<td>LAST STG M2 M4 REL = 0.6651</td>
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<td></td>
</tr>
<tr>
<td>STAGE 1-2 MEANLINE SLOPE = 16.85 DEG BASED ON MID ASPECT-RATIO BLADING</td>
<td></td>
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<td></td>
<td></td>
<td></td>
<td></td>
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<td></td>
<td></td>
<td></td>
<td></td>
</tr>
</tbody>
</table>
the nonexistence of a hub or tip solution satisfying free-vortex conditions. These messages are presented in this section, and their causes are discussed.

(1) INSUFFICIENT ENERGY—This message is caused by the computed turbine exit total temperature being less than zero. It indicates that the turbine specific work requirement is greater than the energy available in the gas. Therefore, either the specified shaft power must be decreased or the specified flow must be increased.

(2) INSUFFICIENT IDEAL ENERGY—This message is caused by the computed ideal energy being more than that available

<table>
<thead>
<tr>
<th>STAGE 1-2 MEANLINE SLOPE</th>
<th>13.98 DEG BASED ON MID ASPECT-RATIO BLADING</th>
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<tr>
<td>POWER FLOW TEMP PRESS SPEED HB DIA HB DIA RADIUS EX AMG CONST CAPAC VISCOSITY LOSS VEL SQ PRESS RATIO COEF RATIO</td>
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<tr>
<td>12900.0 55.50 2640.00 115.10 11400.0 20.00 21.00 0.0000 65.00 53.37 1.302 0.376E-04 0.359 1.200 0.000</td>
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<tr>
<td>STAGES 2 Z EX IT VEN A N E S</td>
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</tr>
<tr>
<td>EXIT HUB DIAMETER = 21.00</td>
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<td>EXIT RADIUS RATIO = 0.8000</td>
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</tr>
<tr>
<td>INLET TIP DIAMETER = 23.04</td>
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<tr>
<td>INLET RADIUS RATIO = 0.8497</td>
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<tr>
<td>LAST STG M1 ABS = 0.8955</td>
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<td>LAST STG M1 REL = 0.4555</td>
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<td>HUB: LAST STG M1 ABS = 0.9106</td>
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<td>LAST STG M1 REL = 0.4681</td>
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<td>LAST STG M2 REL = 0.7507</td>
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<tr>
<td>TIP: LAST STG M1 ABS = 0.7603</td>
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<tr>
<td>LAST STG M1 REL = 0.4844</td>
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<tr>
<td>LAST STG M2 REL = 0.6995</td>
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<table>
<thead>
<tr>
<th>STAGE 1-2 MEANLINE SLOPE</th>
<th>14.19 DEG BASED ON MID ASPECT-RATIO BLADING</th>
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</thead>
<tbody>
<tr>
<td>POWER FLOW TEMP PRESS SPEED HB DIA HB DIA RADIUS EX AMG CONST CAPAC VISCOSITY LOSS VEL SQ PRESS RATIO COEF RATIO</td>
<td></td>
</tr>
<tr>
<td>12900.0 55.50 2640.00 115.10 11400.0 20.00 21.00 0.0000 65.00 53.37 1.302 0.376E-04 0.359 1.200 0.000</td>
<td></td>
</tr>
<tr>
<td>STAGES 2</td>
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<tr>
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<tr>
<td>EXIT HUB DIAMETER = 21.00</td>
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<tr>
<td>TIP: LAST STG M1 ABS = 0.7603</td>
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<tr>
<td>LAST STG M1 REL = 0.4844</td>
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<td>STATOR EXIT ANGLE = 64.84</td>
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<table>
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<th>14.16 DEG BASED ON MID ASPECT-RATIO BLADING</th>
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</thead>
<tbody>
<tr>
<td>POWER FLOW TEMP PRESS SPEED HB DIA HB DIA RADIUS EX AMG CONST CAPAC VISCOSITY LOSS VEL SQ PRESS RATIO COEF RATIO</td>
<td></td>
</tr>
<tr>
<td>0.0 55.50 2640.00 115.10 11400.0 20.00 21.00 0.0000 65.00 53.37 1.302 0.376E-04 0.359 1.200 0.000</td>
<td></td>
</tr>
<tr>
<td>STAGES 2</td>
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<tr>
<td>EXIT TIP DIAMETER = 27.35</td>
<td></td>
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<tr>
<td>EXIT HUB DIAMETER = 21.00</td>
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<td>EXIT RADIUS RATIO = 0.7477</td>
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<td>INLET TIP DIAMETER = 23.04</td>
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<tr>
<td>INLET RADIUS RATIO = 0.8497</td>
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<tr>
<td>SHAEF POWER = 12922.2</td>
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<tr>
<td>LAST STG M1 ABS = 0.7840</td>
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<td>LAST STG M1 REL = 0.5370</td>
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<td>HUB: LAST STG M1 ABS = 0.9106</td>
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<td>TIP: LAST STG M1 ABS = 0.7603</td>
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<td>LAST STG M1 REL = 0.5370</td>
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<td>SHAEF POWER = 12922.2</td>
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<tr>
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</table>

TABLE IV.—Concluded.

TURBINE VELOCITY DIAGRAM ANALYSIS

This is an additional title called for by itit in previous case data.
from an infinite expansion of the gas. It indicates that the computed efficiency is too low to yield a valid solution. Corrective action includes decreasing power, increasing flow, using more stages, or perhaps using a different velocity diagram.

(3) NEGATIVE TEX—This message is caused by the computed turbine exit static temperature being less than zero. It indicates a low value of turbine exit total temperature and/or a high value of turbine exit velocity. Corrective action could be the same as for message (2) or decreasing a too high value of stage exit to average meridional kinetic energy ratio (E).

(4) INSUFFICIENT EXIT AREA—This message is caused by the computed exit area being larger than that available in the turbine. Such a situation can be remedied in many ways, including increasing exit diameter, decreasing stator exit angle, increasing inlet pressure, decreasing mass flow, and increasing stage exit to average meridional kinetic energy ratio.

(5) INSUFFICIENT INLET AREA—This message is caused by the computed inlet area being larger than that available in the turbine. Corrective measures are the same as for message (4).

(6) NEGATIVE T AT STATOR HUB EXIT—This message is caused by the stator-exit hub static temperature being less than zero as a result of a very high absolute velocity. Although a non-free-vortex design (not available from this program) might not result in a similar condition, it is extreme enough to indicate that the mean-section design may require change such as a larger diameter, smaller stator angle, or more stages. This condition does not interrupt the mean section computations.

(7) NEGATIVE T AT ROTOR TIP EXIT—This message is caused by the rotor-exit tip static temperature being less than zero as a result of a very high absolute velocity. Corrective measures are the same as for message (6) above.

References


National Aeronautics and Space Administration
Lewis Research Center, Cleveland, Ohio 44135
August 12, 1991
This mean-diameter flow analysis uses a stage-average velocity diagram as the basis for the computation of efficiency. Input design requirements include power or pressure ratio, flow rate, temperature, pressure, and rotative speed. Turbine designs are generated for any specified number of stages and for any of three types of velocity diagrams (symmetrical, zero exit swirl, or impulse) or for any specified stage swirl split. Exit turning vanes can be included in the design. The program output includes inlet and exit annulus dimensions, exit temperature and pressure, total and static efficiencies, flow angles, and last-stage absolute and relative Mach numbers. This report presents the analysis method and a description of the computer program input and output with sample cases. The analysis and code presented herein are modifications of those described in NASA TN-D-6702. These modifications improve modeling rigor and extend code applicability.