METHOD OF RECERTIFYING A LOADED BEARING MEMBER

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ABSTRACT

The present invention is a method of recertifying a loaded bearing member using ultrasound testing to compensate for different equipment configurations and temperature conditions. The standard frequency F1 of a reference block is determined via an ultrasound tone burst generated by a first pulsed phase locked loop (P2L2) equipment configuration. Once a lock point number s is determined for F1, the reference frequency F1a of the reference block is determined at this lock point number via a second P2L2 equipment configuration to permit an equipment offset compensation factor F01 = ((F1 - F1a)/F1)(1000000) to be determined. Next, a reference frequency F2 of the unloaded bearing member is determined using the second P2L2 equipment configuration and is then compensated for equipment offset errors via the relationship $F_{2c} = F_2 + F_2(F_{01}/1000000)$. A lock point number b is also determined for F2. A resonant frequency F3 is determined for the reference block using a third P2L2 equipment configuration to determine a second offset compensation factor F02 = ((F1 - F3)/F1)(1000000). Next the resonant frequency F4 of the loaded bearing member is measured at lock point number b via the third P2L2 equipment configuration and the bolt load determined by the relationship $load = (-1000000)C1((F_{2c} - F_4)/F_{2c} - F_{02})$, wherein C1 is a factor correlating measured frequency shift to the applied load. Temperature compensation is also performed at each point in the process.

10 Claims, 7 Drawing Sheets
Ultrasonic Thermal Calibration Factor $C_t$

$$y = 5686.1 - 68.550x \quad R^2 = 1.000$$

FIG. 6

Ultrasonic Load Calibration Factor "$C_l$"

$$y = 70.614 - 4.5105x \quad R^2 = 1.000$$

FIG. 4
Shuttle Wheel Bolt Lock Point Spacing vs Load

\[ y = 3987.2 - 123.09x \quad R^2 = 0.994 \]

FIG. 9

Shuttle Wheel Bolt Lock Point Spacing vs Temperature

\[ y = 3.2490e+4 - 1.8982x \quad R^2 = 1.000 \]

FIG. 10
METHOD OF RECERTIFYING A LOADED BEARING MEMBER

ORIGIN OF THE INVENTION

The invention described herein was made by an employee of the United States Government and may be manufactured and used by and for the Government for governmental purposes without the payment of any royalties thereon or therefor.

CROSS-REFERENCE

This application is related to a co-pending application entitled "Method of Recertifying a Loaded Bearing Member Using a Phase Point", Ser. No. 07/720,153, filed Jun. 19, 1991.

BACKGROUND OF THE INVENTION

1. Technical Field

The present invention relates generally to material testing and more particularly to a method of recertifying the load on a bearing member via ultrasound techniques.

2. Discussion of the Related Art

The pulsed phase locked loop strain monitor, or P2L2, is a device which can be utilized to nondestructively measure the load in a bearing member such as a bolt, connector, etc. The P2L2, described in U.S. Pat. No. 4,363,242 to Heyman, measures acoustic phase changes and displays frequency changes which are indicative of changes in the load of the bearing member. By only determining load changes, the P2L2 requires that the ultrasonic sensor be kept on the bearing member during tightening or other load applications. Measurement of the load in a tightened bearing member, i.e., load recertification, is not possible because temperature variations as well as substitutions of various P2L2 elements such as transducers, cables, ultrasonic bonding, coupling, the P2L2 member itself, etc. are not corrected for a subsequent, after load measurement. In addition, it is often difficult to return to the same frequency point to correctly determine a subsequent frequency shift, thereby resulting in incorrect load determination.

Other non-destructive load testing systems include using an ultrasonic monitor to measure the time of flight of ultrasonic waves through a bolt. This measured time of flight is indicative of bolt loading. Such time of flight systems are harder to use and generally less accurate than P2L2 systems because the correct threshold trigger point must be selected to give the correct load measurement. In addition, load recertification of an already tightened bearing member is difficult for the reasons discussed above in connection with the P2L2 system.

OBJECTS OF THE INVENTION

It is accordingly an object of the present invention to permit load recertification of a loaded bearing member.

It is another object of the present invention to accomplish the foregoing object while accounting for changes in temperature.

It is a further object of the present invention to accomplish the foregoing objects while permitting substitutions of various P2L2 system components and recertification subsystem components.

It is another object of the present invention to avoid the difficulties and incorrect load determination associated with returning to a frequency peak lock point number after a period of time has elapsed.

It is yet another object of the present invention to accomplish the foregoing objects non-destructively.

It is a further object of the present invention to accomplish the foregoing objects in a simple manner.

Additional objects and advantages of the present invention are apparent from the drawings and specifications.

SUMMARY OF THE INVENTION

The foregoing and additional objects are achieved by a method for recertifying the load of a bearing member according to the present invention. The standard frequency F1 of a reference block is determined via an ultrasonic tone burst generated by a first pulsed phase locked loop (P2L2) equipment configuration. Once a lock point number is determined for F1, the reference frequency F1a of the reference block is determined at this lock point number via a second P2L2 equipment configuration to permit an equipment offset compensation factor F01=((F1-F1a)/F1) (1000000) to be determined. Next, a reference frequency F2 of the unloaded bearing member is determined using the second P2L2 equipment configuration and is then compensated for equipment offset errors via the relationship F2c=F2+F2(F01)/1000000. A lock point number b is also determined for F2. A resonant frequency F3 is determined for the reference block using a third P2L2 equipment configuration to determine a second offset compensation factor F02=((F1-F3)/F1) (1000000). Next the resonant frequency F4 of the loaded bearing member is measured at lock point number b via the third P2L2 equipment configuration and the bolt load determined by the relationship bolt load=−(−1000000-JC1)((F2c−F4)/F2c)−Fo2), wherein C1 is a factor correlating measured frequency shift to the applied load. Temperature compensation is also performed at each point in the process.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is a schematic diagram of the constituent elements of a prior art pulsed phase locked loop strain monitor;

FIG. 2 is a schematic diagram of a prior art pulsed phase locked loop strain monitor and associated hardware;

FIG. 3 is a schematic diagram of a pulsed phase locked loop equipment configuration according to the present invention;

FIG. 4 is a graph showing the correlation of the frequency shift ΔF/F with bolt load;

FIG. 5 is a graph schematically depicting a typical ultrasonic frequency spectrum of a bearing member showing the lock point for a loaded and unloaded bolt and the load-induced frequency shift of a typical lock point;

FIG. 6 is a graph showing the correlation of the frequency shift with temperature;

FIG. 7 is a graph depicting the frequency spacing of successive lock points for a loaded and unloaded bolt;

FIGS. 8a and 8b show the P2L2 output pulse and phase signals for unlocked and locked conditions as displayed on an oscilloscope;

FIG. 9 is a graph showing the relationship between bolt load and lock point spacing; and

FIG. 10 is a graph showing the relationship between lock point spacing and temperature.
DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENT

Referring to FIG. 1, a pulse phase locked loop strain monitor or P2L2 10 is schematically shown. The P2L2 is described in greater detail in U.S. Pat. No. 4,363,242 to Heyman, the specification of which is hereby incorporated by reference. The P2L2 measures acoustic phase change and reads out corresponding frequency changes. In general terms, the RF output of a voltage controlled oscillator (VCO) 12 is periodically gated by gate 14 and then amplified and transmitted via amplifier transmitter 16 to an ultrasonic transducer 18. Transducer 18 is affixed to an end of test material 20 via an appropriate couplant 19 such as water, glycerin, light machine oil, etc. and produces an acoustic tone burst or sound wave pulse which propagates in bearing member 20. The produced acoustic signal may have any ultrasonic frequency near the center of the operating frequency bandwidth of the compound resonator formed by the transducer bonded to the load bearing member. Also, test material 20 may be a load bearing member such as a bolt or other load bearing component of any geometrical configuration.

The generated tone burst or sound wave pulse is reflected by the far end of test material 20 back to transducer 18 and converted to an electrical signal by the transducer. The signal is gated by gate 22, amplified by amplifier receiver 24, and sent to mixer phase detector 26. Mixer phase detector 26 also receives the output of VCO 12 and produces a DC signal proportional to the phase difference between these two inputs. Logic timing system 28 controls the gating time of gates 14 and 22 in response to a signal received from VCO 12. Logic timing system 28 also controls the time at which a phase point of the phase difference signal is sampled and held by sample/hold circuit 30. This sampled signal is then appropriately conditioned by signal conditioner 32 and sent to VCO 12 to control the VCO output frequency. The P2L2 locks onto resonant frequencies which correspond to quadrature which represents a phase delay of \( 2\pi N + \phi_0 \), where \( N \) is an integer and \( \phi_0 \) is typically 90°.

Once the frequency \( F \) is locked at a particular lock point, any deviation in the propagation length or sound speed of the test material 20 results in a frequency change \( \Delta F \) needed to maintain a fixed phase condition. Thus, the P2L2 10 measures frequency changes to indicate load changes, i.e., tension or compression, which change the sound velocity and propagation length. This frequency change relationship \( \Delta F/F \) resulting from the load change is interpreted via a load-member-specific ultrasonic load calibration factor \( C_1 \), which is defined in units of lb/psm to indicate pounds of load change per parts per million of normalized frequency shift. The factor \( C_1 \) is determined by obtaining frequency changes resulting from loads for the particular bearing member and obtaining a polynomial function which represents an acceptable curve fit for the obtained data. For simplicity, a first order linear polynomial is generated and the x-coefficient representing the slope of the curve is expressed as \( C_1 \). Then when the load is changed, the normalized frequency shift is multiplied by the \( C_1 \) factor to determine the load change. See FIG. 4.

If the load is the only parameter changed, then the displayed frequency change is indicative solely of this load change. However, changes in other parameters such as equipment and temperature may also affect the frequency changes, thereby contaminating the subsequent load change determination. Compensation must accordingly be made for these other parameter changes. Changes in the measurement equipment configuration typically occur over the period of time between the initial and subsequent measurements as specific components are replaced due to loss, failure or interchange with other configurations. Also, temperature fluctuations can have significant effects, e.g., a 1° F. change can result in a frequency change which would incorrectly indicate a 350 lb. load change in a shuttle wheel bolt.

As shown in FIG. 2, an oscilloscope 34 and a frequency counter 36 are connected to P2L2 10 to provide a respective visual display and an electronic readout of the VCO 12 frequency (FIG. 1). Test material 20, e.g., a bolt, is passed through a load cell 38 and has a nut 40 threaded on the end opposite transducer 18 which is tightened to induce a load. Alternatively, the transducer can be coupled to the nut end of the bolt. Load cell 38 duplicates the actual field thickness where the bolt is to be installed. A voltmeter 42 reads the voltage induced in load cell 38 by the tightening of nut 40. Alternatively, the load can be measured by a hydraulic or other type of indicator. The outputs of voltmeter 42 and frequency counter 36 are read by computer 44 as nut 40 is tightened. This data is fed to an appropriate printer/plotter 46 for display.

The method and system for testing load bearing members according to the present invention will now be described with particular reference to FIGS. 3 and 4. Prior to performing any load measurement of bearing member 20, the particular equipment configurations, e.g., P2L2 10, transducer 18, connecting cable 48, couplant 19, etc., for detecting the VCO frequency changes is selected. This configuration, designed configuration \( A \), is connected to reference or calibration block 50 as indicated by the dotted line indicating cable 48 in FIG. 3. In addition, a thermometer 52 is connected to reference block 50 via cable 54, as indicated by the dotted line 54 in FIG. 3. Of course, the thermometer may be hand-held and read. The following standard measurements and determinations are obtained for reference block 50:

1. measuring the standard lock point frequency \( F_1 \) (Hz) via the particular transducer 18, cable 48, couplant, and P2L2 10 of configuration \( A \);
2. measuring the temperature \( T_1 \) (°F) via thermometer 52 and cable 54;
3. determining the calibration standard lock point number \( s \); and
4. determining the reference block ultrasonic thermal calibration factor \( C_{tr} \) (ppm/°F).

Step (3) involves measuring the frequency separation or spacing between two successive lock points at temperature \( T_1 \) and then dividing \( F_1 \) by this measured spacing. Each lock point is a mechanical resonant frequency of the calibration block 50. In FIG. 5 the general case is illustrated wherein the lock point number is represented by \( m \) and the mechanical resonant frequencies are represented by peaks. The fundamental resonant frequency corresponds to \( m = 1 \) and the \( m \)th harmonic resonant frequency is represented by the \( m \)th peak in FIG. 5. The value of \( m \) is generally equal to \( F/F_{m} + F_{m} \) and \( s \) is specifically equal to \( F_1/(F_{s+1} - F_{s}) \), wherein \( F_1 \) and \( F_{s+1} \) are any two successive harmonic frequencies. The determination of the lock point number \( m \) or \( s \) allows the P2L2 to return to...
the same frequency peak after an induced load to determine the true load-induced frequency shift. The same lock point number must be used because often this frequency shift is greater than the harmonic interval or lock point spacing and accordingly merely using the frequency F1 to recertify the load could result in a significant misreading of the frequency change. The lock point number is optimally chosen so that the frequencies lie near the center of the operating frequency bandwidth of the compound resonator formed by the transducer 18 bonded to the load bearing member 20 by the couplant 19.

The determination of the material specific Ctr in step (4) involves use of an experimentally derived relationship between the change in resonant or harmonic frequency and temperature, i.e., a shift in the lock point s and the change in temperature, as generally represented in FIG. 6. For example, the reference block may be placed in an oven and its frequency observed at various temperatures. Like CI, Ctr represents the slope of a linear or other polynomial function which best fits the generated data curve.

Next, another equipment configuration comprising at least one element, and as many as all elements, different from the corresponding elements in configuration A is selected for determining the VCO frequency changes. This other configuration, designated configuration B, is also connected to reference block 50 as indicated by the dotted lines in FIG. 3 prior to performing any load measurement of bearing member 20. If any element is changed in the configuration, the true propagation path length also changes, resulting in all of the lock point frequencies being shifted by approximately the same amount. Similarly, a temperature change can also shift the lock point frequencies. Equipment offset errors are obtained for configuration B by the following steps:

(5) measuring the reference lock block point frequency F1a (Hz) via the particular transducer 18, cable 48, couplant and P2L2 10 of configuration B using lock point number s;

(6) measuring the reference block temperature T1a (°F) via thermometer 52 and cable 54;

(7) compensating F1a for temperature difference T1−T1a using Ctr such that:

\[ F_{1ac} = F_{1a} - Ctr(T1−T1a)F1/1000000, \]

wherein F1ac is compensated F1a; and

(8) determining the dimensionless offset compensation factor F01 expressed in parts of frequency shift per million parts of operating frequency:

\[ F01 = (F1 − F1ac)/F11000000, \]

(9) measuring the reference lock point frequency F2 (Hz) of the unloaded bolt (bearing member 20) via the particular transducer 18, cable 48, couplant and P2L2 10 of configuration B;

(10) measuring the unloaded bolt reference temperature T2 (°F) via thermometer 52 and cable 54;

(11) determining the bolt lock point number b by measuring the frequency separation or spacing between two lock points and dividing this spacing into F2 as described above in reference to lock point number s;

(12) compensating the unloaded bolt reference frequency for equipment offset errors with the previously determined reference block offset compensation factor F01 (step 8) by

\[ F_{2c} = F2 + F01/F11000000, \]

wherein F2c is the compensated frequency;

(13) determining the bolt ultrasonic load calibration factor CI (lb/ppm) as discussed above; and

(14) determining the material specific, bolt ultrasonic thermal calibration factor Ct (ppm/°F) using the experimentally derived relationship between the normalized shift of lock point frequency number b and the change in temperature as generally shown in FIG. 6 to compensate for any subsequent temperature deviation from the initial bolt temperature T2. Ct may be determined by the same method as Ctr.

Once all of the measurements and determinations are made in steps (1)–(14), the bearing member 20, e.g., a bolt, is installed under load (preloaded) in a field condition rather than in the approximating load cell 38. In all likelihood, the tester would prefer the widest possible latitude in selecting the particular equipment configuration to determine the load indicating VCO frequency changes at the particular test time due to equipment repair, replacement and availability. A new configuration C having at least one element and possibly all elements, different from the corresponding elements in configurations A and B, is selected. Also, the effects of a different temperature at test time from the initial measurements must be taken into account. As stated previously, it is paramount to know which lock point number of the bearing member is being used for the measurement in order to assess the load on the load bearing member with any equipment and temperature changes. Configuration C is accordingly connected to reference block 50 to obtain the following measurements and error corrections using lock point number s:

(15) measuring the reference block lock point frequency F3 (Hz) at lock point number s via the particular transducer 18, cable 48, couplant and P2L2 10 of configuration C;

(16) measuring the reference block temperature T3 (°F) via thermometer 52 and cable 54;

(17) compensating F3 for temperature difference T1−T3 using Ctr by the following:

\[ F_{3c} = F3 - Ctr(T1−T3)F3/1000000, \]

wherein F3c is the temperature compensated frequency F3; and

(18) calculating the offset compensation F02 by

\[ F02 = (F3 − F3c)/F31000000. \]

If no compensation is made for the temperature-induced frequency change, F3 is substituted for F3c in the preceding equation. Finally, the loaded bolt at the test site is connected to configuration C and the following steps taken:

(19) measuring the bolt frequency F4 (Hz) at lock point b via the particular transducer 28, cable 48, couplant and P2L2 10 of configuration C;
F4; and
point number by a subsequently measured spacing be-
substituted for F2c and F4c in the preceding equation.
In practice, numerous bolts could be measured at one
time during steps (5)–(14) using equipment configura-
tion B to determine the respective compensated bolt
reference frequencies and temperatures. Steps (1)–(4)
related to the calibration reference block 50 need only
be performed once regardless of the number of bolts
tested. Steps (15)–(22) are performed as desired to
recertify this bolt load with the particular contempora-
aneous equipment configuration C and temperature T3.

As noted previously, both the lock point number s for
reference block 50 and lock point number b for bearing
member 20 can be determined by measuring the fre-
quency spacing between respective successive lock
points at a particular temperature and then dividing this
frequency spacing into the frequency of the particular
element at this temperature. The subsequent lock point
frequencies are then determined by multiplying the lock
point number by a subsequently measured spacing be-
tween two successive lock points.

Another method will now be described to allow one
to subsequently obtain the same lock point in order to
perform recertification of a load by the same equip-
ment configuration. This method locks at the same phase
point on the acoustic signal done visually using an oscil-
loscope and duplicating the sample/hold
setting (S/H) setting N on the P2L2. As shown in FIGS. 5 and 7, a frequency
F no defines an amplitude or resonant peak wherein
FNA=mV/2L, wherein m is the harmonic number, V is
the velocity of the acoustic tone burst through the bear-
ing member, and L is the length of the bearing member
through which the acoustic tone burst travels. The
number of wavelengths λ in the bolt path is 2L/λ,
wherein 2L is the round-trip or two-way length of the
propagation path in the bolt. The number of wave-
lengths is thus equal to 2L/λ, which is equal to Fm/T, wherein T is the time required
for propagation along the bolt and back. Thus, N repre-
sents a specific point in phase that is determined by the
bolt length, frequency and velocity.

Referring to FIGS. 8a and 8b, in operation a certain
P2L2 S/H setting N is selected as shown in FIG. 8a,
then the P2L2 is locked so that it is in quadrature as
shown in FIG. 8b to drive the phase signal to 90°, ini-
itially for a no load condition and the wave number w
in the tone burst where the S/H marker appears is
noted. For example, w is the integral number of the
third wave counting from left to right in FIG. 8b. The
initial temperature, S/H setting N and wave number w
are recorded. When the bearing member is tightened to
induce a load with the P2L2 locked as shown in FIG.
8b, the frequency and the phase sample point both will
shift by an amount equal to the shift in time caused by
the bolt tension. If, after the bolt is loaded, the measure-
ment system is removed from the bolt causing loss of
the original P2L2 lock and then the system is reinstated
to recertify load later on, re-locking without changing
any of the P2L2 settings may result in relock at a dif-
ferent lock point and will imply an incorrect load. To
avoid any misreading, the S/H setting N on the P2L2 is
duplicated and the P2L2 VCO frequency is adjusted
with the P2L2 unlocked until the S/H marker is at the
same wave number w on the acoustic signal as during
the initial measurement. The P2L2 is then locked at the
wth wave to provide a valid measurement of the fre-
quency for determination of bolt load using this fre-
quency for the final frequency and the particular CI. In
addition, the final temperature is measured and any
temperature-induced frequency change compensated
for using this temperature, the initial temperature, and a
thermal calibration factor as discussed above.

An alternate approach to load recertification will now
be explained. The previous two methods measure the
frequency shift of a lock point, i.e., ΔF, to determine
the load as a function of ΔF/F. This approach, on the
other hand, measures the change in frequency spacing
between two lock points as a function of bolt load. As
illustrated in FIGS. 7 and 9, as bolt load increases the
spacing between two successive lock points decreases.
Lock point spacing is measured using an oscilloscope to
observe the P2L2 signals by first locking to a lock point
near the center of the operating frequency bandwidth
of the compound resonator formed by the transducer
bonded to the load bearing member by the couplant,
recording this original frequency, then unlocking the
P2L2, adjusting the VCO frequency until 360° (2π)
of phase shift has occurred, adjusting the S/H position
back to the original phase point on the ultrasonic signal,
then re-locking the P2L2 and subtracting the new fre-
quency from the original frequency to determine spac-
ing. After loading, the above steps are repeated to deter-
mine a subsequent lock point spacing.

Referring to FIG. 9, the lock point spacing changes
as to a function of load. A load calibration factor Cs
can be determined as discussed previously for Ct,
wherein Cs is a part of a polynomial which represents
an acceptable curve fit for the previously compiled load/frequency spacing data. Thus, a determination of
the frequency spacing leads to a load determination.

Referring now to FIG. 10, the lock point spacing also
changes with temperature. Accordingly, a thermal cali-
bration factor Cst is determined as discussed previously
for Ct and Ctr, wherein Cst is a part of a polynomial
which represents an acceptable curve fit for the previ-
ously compiled temperature/frequency spacing data.
Thus, if an initial temperature reading is taken substan-
tially contemporaneously with the original frequency
spacing determination and a subsequent temperature
reading is taken substantially contemporaneously with
the subsequent frequency spacing determination, then
a temperature compensation can be performed on the
reading as discussed previously.

The determinations and calculations of the above
described values can be carried out by computer using
appropriate software embodying the meth-
odologies described above.
Many modifications, improvements and substitutions will be apparent to the skilled artisan without departing from the spirit and scope of the present invention as described herein and defined in the following claims. What is claimed is:

1. A method of rectifying a load on a bearing member having a selected harmonic resonant frequency of the bearing member including the steps of:

(a) performing the following steps on a reference block:

(i) applying an ultrasonic tone burst to the reference block via a first transducer of a first P2L2 equipment configuration to determine the standard frequency \( F_1 \) of the reference block;

(ii) determining a lock point number \( s \) indicative of a selected harmonic resonant frequency of the reference block;

(iii) applying an ultrasonic tone burst to the reference block via a second transducer of a second P2L2 equipment configuration comprising at least one element different from the elements of the first equipment configuration, to determine the resonant frequency \( F_{1a} \) of the reference block at lock point \( s \);

(iv) determining a first dimensionless offset compensation factor \( F_{01} \) by the relationship

\[
F_{01} = \frac{F_1 - F_{1a}}{F_1} \times 10^6
\]

(b) performing the following steps on the unloaded bearing member:

(i) applying an ultrasonic tone burst to the unloaded bearing member using the second equipment configuration to determine the reference frequency \( F_2 \) of the unloaded bearing member;

(ii) determining a lock point number \( b \) indicative of a selected harmonic resonant frequency of the bearing member;

(iii) compensating the unloaded bearing member frequency \( F_2 \) for equipment offset errors between the first and second equipment configurations by the relationship

\[
F_{2c} = F_2 + \frac{F_2(F_{01})}{10^6}
\]

(c) performing the following steps on the reference block:

(i) applying an ultrasonic tone burst to the reference block via a third equipment configuration having at least one element different from the first and second equipment configurations, to determine a resonant frequency \( F_3 \) at lock point number \( s \) of the reference block;

(ii) determining a second dimensionless offset compensation factor \( F_{02} \) by the relationship

\[
F_{02} = \frac{F_1 - F_3}{F_1} \times 10^6
\]

(d) performing the following steps on the bearing member after a load is applied thereto:

(i) applying an ultrasonic tone burst to the loaded bearing member via the third equipment configuration to determine a resonant frequency \( F_4 \) at lock point number \( b \) of the loaded bearing member;

(ii) determining the bolt load equal to \((-1000000)\)

\[
C_1((F_2c - F_4)/F_2c) - F_{02}
\]

wherein \( C_1 \) is a bearing member calibration factor correlating the measured frequency shift to the applied load.

2. The method according to claim 1, wherein step (a)(iii) comprises determining the difference between two successive resonant frequencies of the reference block and multiplying this by \( s \) to obtain a frequency, adjusting the P2L2 to this frequency and locking the P2L2 to measure \( F_{1a} \).

3. The method according to claim 1, wherein step (b)(ii) comprises determining the difference between two successive resonant frequencies of the bearing member and dividing this difference into \( F_2 \) to obtain \( b \).

4. The method according to claim 1, further comprising the steps of measuring the temperature \( T_1 \) of the reference block substantially contemporaneously with step (a)(i);

measuring the temperature \( T_{1a} \) of the reference block substantially contemporaneously with step (a)(iii);

and compensating frequency \( F_{1a} \) for any change in temperature via the relationship \( F_{1a} = C_{tr} \times F_1 - C_{t1a} \times F_1 \) prior to step (a)(iv), wherein \( C_{tr} \) is a determined thermal calibration factor correlating temperature change and resonant frequency change in the reference block.

5. The method according to claim 4, further comprising the steps of measuring the temperature \( T_2 \) of the unloaded bearing member substantially contemporaneously with step (b)(i);

measuring the temperature \( T_4 \) of the loaded bearing member substantially contemporaneously with step (d)(i); and

compensating frequency \( F_{4} \) for any change in temperature via the relationship \( F_4 = C_{tr} \times F_1 - C_{t4} \times F_1 \) prior to step (d)(ii), wherein \( C_{tr} \) is a determined calibration factor correlating temperature change and resonant frequency change in the bearing member.

6. The method according to claim 1, further comprising the steps of measuring the temperature \( T_2 \) of the unloaded bearing member substantially contemporaneously with step (b)(i);

measuring the temperature \( T_4 \) of the loaded bearing member substantially contemporaneously with step (d)(i); and

compensating frequency \( F_4 \) for any change in temperature via the relationship \( F_4 = C_{tr} \times F_1 - C_{t4} \times F_1 \) prior to step (d)(ii), wherein \( C_{tr} \) is a determined calibration factor correlating temperature change and resonant frequency change in the bearing member.

7. The method according to claim 1, further comprising the steps of measuring the temperature \( T_1 \) of the reference block substantially contemporaneously with step (a)(i);

measuring the temperature \( T_3 \) of the reference block substantially contemporaneously with step (c)(i); and

compensating frequency \( F_3 \) for any change in temperature via the relationship \( F_3 = C_{tr} \times F_1 - C_{t3} \times F_1 \) prior to step (c)(ii), wherein \( C_{tr} \) is a determined thermal calibration factor correlating temperature change and resonant frequency changes in the reference block.

8. A method of rectifying a load on a bearing member comprising the steps of:
(a) applying an ultrasonic tone burst to the bearing member in an unloaded condition;
(b) determining the spacing between two successive harmonic resonant frequencies of the unloaded bearing member;
(c) applying an ultrasonic tone burst to the bearing member in an undetermined loaded condition;
(d) determining the spacing between two successive harmonic resonant frequencies of the loaded bearing member; and
(e) computing the load on the bearing member based on the difference in spacing between the unloaded and loaded conditions.

9. The method according to claim 8, further comprising the steps of:

(f) measuring the temperature of the unloaded bearing member substantially contemporaneously with step (b);
(g) measuring the temperature of the loaded bearing member substantially contemporaneously with step (d); and
(h) compensating for any temperature-induced change in the spacing using a predetermined thermal correction factor and the difference between the temperatures measured in steps (f) and (g).

10. The method according to claim 8, wherein the load on the bearing member is determined by multiplying the difference in spacing between the unloaded and loaded condition by a predetermined load calibration factor.

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