A METHOD FOR DETERMINING THE COLUMN CURVE
FROM TESTS OF COLUMNS WITH EQUAL RESTRAINTS
AGAINST ROTATION ON THE ENDS
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A METHOD FOR DETERMINING THE COLUMN CURVE FROM TESTS OF COLUMNS WITH EQUAL RESTRAINTS AGAINST ROTATION ON THE ENDS

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SUMMARY

The results are presented of a theoretical study for the determination of the column curve from tests of column specimens having ends equally restrained against rotation. The theory of this problem is studied and a curve is shown relating the fixity coefficient \( c \) to the critical load, the length of the column, and the magnitude of the elastic restraint. A method of using this curve for the determination of the column curve for columns with pin ends from tests of columns with elastically restrained ends is presented. The results of the method as applied to a series of tests on thin-strip columns of stainless steel are also given.

INTRODUCTION

It is fairly common practice for engineers to assume that a given fixture or restraining member at the end of a column provides a column fixity coefficient \( c \) that is constant for columns of different lengths and sizes of column cross sections. In reality, however, the fixity coefficient depends upon both the dimensions of the column and the restraint offered by the restraining member; therefore, a more correct procedure, theoretically, would be to assume that a given fixture or restraining member at the end of the column provides, rather than a given fixity coefficient, a restraining stiffness against rotation that is independent of the dimensions of the column. A method for determining the column curve for thin sheet
material that is based upon considerations of the restraining stiffness and that shows promise of successful development involves mounting each end of a small column specimen in special end fixtures of the general type shown in figure 1. The specimen with end fixtures attached is then loaded by axially compressing the assembly until the specimen fails by instability as an ordinary column.

The special end fixtures on the column specimen in figure 1 tend to produce a condition approaching that of completely fixed ends. Actually, the elasticity of the clamping fixtures and of the specimen does not permit completely fixed ends and, as a consequence, the column fixity coefficient \( c \), which is 1 for pin ends and 4 for fixed ends, will be slightly less than 4.

The method of analyzing test data to obtain the column curve for any thin sheet material, tested in the manner described, is presented herein. Column curves that have been determined in this manner for a series of stainless-steel specimens are also presented.

DETERMINATION OF COLUMN CURVE FOR \( c = 1 \)

The column curve that is usually plotted for any material is the curve for pin-ended columns, \( c = 1 \). In order to obtain the column curve from tests employing the end fixtures shown in figure 1, the value of the column fixity coefficient \( c \) must be determined. This coefficient \( c \) can be determined by use of the curve of figure 2 provided the length of the column \( L \), the experimental critical buckling load of the column \( P_{cr} \), and the restraining stiffness of the end fixtures \( m \) (assumed to be the same at each end), are known. The curve shown in figure 2 is the graph of the equation

\[
\frac{P_{cr}L}{m} = \frac{\pi}{\sqrt{c}} \tan \left( \frac{\pi}{\sqrt{c}} \right)
\]

which has been determined through consideration of the stability of a column having ends equally restrained against rotation and is a special case of the general solution given by equation (A-13) in the appendix.
After the value of \( c \) has been found, the effective value of the slenderness ratio \( \frac{l}{\rho} \) for the fixity coefficient \( c = 1 \) can be determined by use of the following relation:

\[
\left( \frac{l}{\rho} \right)_{\text{eff}} = \frac{1}{\sqrt{c}} \left( \frac{l}{\rho} \right)_{\text{exp}}
\]

where

\[
\left( \frac{l}{\rho} \right)_{\text{eff}} \quad \text{effective slenderness ratio for } c = 1
\]

\[
\left( \frac{l}{\rho} \right)_{\text{exp}} \quad \text{actual slenderness ratio of tested column}
\]

and

\( \rho \quad \text{radius of gyration of column} \)

The column curve for pin ends is obtained by plotting \( \frac{P_{cr}}{A} \) against \( \left( \frac{l}{\rho} \right)_{\text{eff}} \) for the specimens tested, where \( A \) is the cross-sectional area of the specimen.

**EVALUATION OF RESTRAINT \( m \)**

Before the fixity coefficient \( c \) can be evaluated by the procedure given in the foregoing section, the value of the restraint \( m \) of each end fixture must be known. In the case of the end fixture of figure 1, the specimen is so narrow compared to the width of the fixture that probably only a part of the fixture is elastically restraining the rotation of the end of the specimen. It is, therefore, likely that the restraint \( m \) depends on the width of the specimen and should be evaluated for each different width. The value of \( m \) for any group of column specimens of the same width and thickness can be determined from the test data for the longer specimens of the group, because the critical compressive stress for these specimens is low and lies within the elastic range where
the effective modulus \( \overline{E} \) is equal to Young's modulus \( E \). For these longer test specimens, all quantities in the Euler column formula are known except \( c \). Therefore

\[
c = \frac{P_{cr}L^3}{n^2EI}
\]  

(1)

The value of \( \frac{P_{cr}L}{m} \) corresponding to this value of \( c \) may be found from the graph of figure 2; then \( m \) can be computed because \( P_{cr} \) and \( L \) are known. When \( m \) has been determined, the evaluation of \( c \) for the columns of shorter lengths can be made as described in the preceding section.

In order to retain the degree of accuracy required in computations, the curve of figure 2 should be plotted to an enlarged scale. As an aid in preparing such a curve, a set of coordinates of the curve is given in table 1.

RESULTS OF TESTS OF STAINLESS-STEEL COLUMNS

OF THIN SHEET MATERIAL

The method for determining the column curve as described in this report has been applied to eight series designated A, B, C, etc. of stainless-steel column specimens. The heat treatment and condition of loading for each series is given in table 2. The chemical analysis for each series is as follows:
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<th>Series F</th>
<th>G and H</th>
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</tbody>
</table>

The column tests were made of flat strips with the type of fixture shown in figure 1. The specimens were loaded in a standard 100,000-pound-capacity hydraulic testing machine. The heads of the testing machine were set parallel and the specimens were centered to produce as nearly an axial load condition as possible. The testing machine was of the two-screw type and the column specimens were so positioned that failure would occur in the plane of the screws of the machine, as the stiffness of the testing machine is greatest in this plane.

The values of E were obtained from stress–strain tests of column specimens from each series of columns tested. A tensile modulus was used in the analysis of several series because it was not possible to determine the compressive modulus. A summary of the values of E and of restraint values m found for the different series is given in Table 2.

Table 3 shows the computations necessary to analyze series A by the method outlined previously in this report; the column curve for series A is shown in figure 3. Graphs of the results of the tests, in which critical stress has been plotted against slenderness ratio for c = 1, are shown in figures 3 to 10.
EFFECT OF ERRORS IN \( E \) ON DERIVED COLUMN CURVE

The application of the procedures outlined in this report to the data obtained from the tests of the stain-
less-steel columns revealed that the accuracy of the re-
sulting column curve depended on the choice of the value
of \( E \) used in the evaluation of the restraint \( m \). A
study was therefore made to determine the effect that
errors in \( E \) would have on the final column curve. The
results of this study revealed that (see fig. 11):

1. At any value of \((L/\rho)_{\text{eff}}\) the stress given by a
derived column curve based upon a value of \( E \) smaller
then the correct value will be in error on the conserva-
tive or low side as compared with the curve derived by
using the correct value of \( E \). For a given percentage
error in \( E \), a low value of \( E \) will give, over the most
practical range of \((L/\rho)_{\text{eff}}\), a smaller percentage error
in column stress than will a correspondingly high value
of \( E \).

2. The maximum percentage error in stress given by
a column curve derived by using an incorrect value of
\( E \) is greater than the percentage error in \( E \) (see
fig. 11). It is therefore important that the compression
modulus for the material of the column specimens be ac-
curately known if a column curve is derived by the method
given in this report.

3. If the correct value of \( E \) is unknown and a com-
monly accepted value of \( E \) is assumed for use in the com-
putations, the analysis should reveal the presence of any
unreasonably large errors in this choice of \( E \). If the
value of \( E \) is too large, the derived column curve will
lie above the Euler curve for this value of \( E \) in the
range of intermediate column lengths (see fig. 12). If
the value of \( E \) used in the evaluation of \( m \) from the
test data for the longer specimens of the group is suffi-
ciently less than the correct value of \( E \), the computed
value of the fixity coefficient, as given by equation (1),
will be greater than \( 4 \).
Consideration of the foregoing facts indicates that, if the value of $E$ is uncertain but is known to lie within a small range, the lowest value in that range would be the most suitable value to select for use in computing the column curve of the material.

Langley Memorial Aeronautical Laboratory, National Advisory Committee for Aeronautics, Langley Field, Va., June 10, 1943.

APPENDIX

THE STABILITY OF A COLUMN HAVING ELASTICALLY RESTRAINED ENDS

Timoshenko has obtained the expressions for the rotations $\theta_a$ and $\theta_b$ at the ends of a member subjected to compressive forces $P$ and end moments $M_a$ and $M_b$ (shown in fig. 13) by solving the differential equation expressing the equilibrium of the bent member. These expressions, given as equation (25) on page 13 of reference 1, are:

\[ \theta_a = \frac{M_a LG}{3EI} + \frac{M_b LF}{6EI} \]  \hspace{1cm} (A-1)

\[ \theta_b = \frac{M_b LG}{3EI} + \frac{M_a LF}{6EI} \] \hspace{1cm} (A-2)

where

\[ F = \varphi(u) = \frac{3}{u} \left( \frac{1}{\sin 2u} - \frac{1}{2u} \right) \] \hspace{1cm} (A-3)

\[ g = \psi(u) = \frac{3}{2u} \left( \frac{1}{2u} - \frac{1}{\tan 2u} \right) \] \hspace{1cm} (A-4)

\[ u = \frac{kL}{2} \] \hspace{1cm} (A-5)
\[ k = \sqrt{\frac{P}{EI}} \]  

(A-6)

In the foregoing equations the symbols have the following meanings:

\( \theta_a, \theta_b \) rotations of the tangents to deflection curve at the left and right ends, respectively, of the member; positive as shown in figure 13

\( M_a, M_b \) applied moments at the left and right ends, respectively, of the member as shown in figure 13

\( L \) length of the member

\( \bar{E} \) effective modulus of elasticity

\( I \) moment of inertia of the column cross section

\( P \) axial load acting on the member

If the moments \( M_a \) and \( M_b \) in equations (A-1) and (A-2) arise as a result of rotation of the restraining elements that oppose the deflection of the member when it bends and if, in addition, the restraining moments are proportional to the rotations of the restraining elements, respectively, the end moments acting on the member can be written

\[ M_a = -m_a \theta_a \]  

(A-7)

\[ M_b = -m_b \theta_b \]  

(A-8)

where \( m_a \) and \( m_b \) are the moments necessary to rotate the restraining elements at the left and right ends of the column, respectively, through one radian.

Substitution of these values of \( M_a \) and \( M_b \) in equations (A-1) and (A-2) gives

\[ (1 + \frac{G\epsilon_a}{3}) \theta_a + \left( \frac{F\epsilon_b}{6} \right) \theta_b = 0 \]  

(A-9)
\[
\left(\frac{F \varepsilon_a}{6}\right) \theta_a + \left(1 + \frac{G \varepsilon_b}{3}\right) \theta_b = 0 \quad (A-10)
\]

where
\[
\varepsilon_a = \frac{m_a L}{EI} \quad (A-11)
\]
\[
\varepsilon_b = \frac{m_b L}{EI} \quad (A-12)
\]

The quantities \( \varepsilon_a \) and \( \varepsilon_b \) are the "restraint coefficient" at the left and right ends, respectively, of the compression member.

Equations \((A-9)\) and \((A-10)\) are linear simultaneous equations which can be satisfied by taking \( \theta_a = \theta_b = 0 \). The deflection at each point on the member is then zero and the straight form of equilibrium of the member is obtained. The buckled form of equilibrium of the member becomes possible only if equations \((A-9)\) and \((A-10)\) yield for \( \theta_a \) and \( \theta_b \) solutions different from zero, which requires that the determinant of these equations become zero, or
\[
\left(1 + \frac{G}{3} \varepsilon_a\right)\left(1 + \frac{G}{3} \varepsilon_b\right) - \left(\frac{F}{6} \varepsilon_a\right)\left(\frac{F}{6} \varepsilon_b\right) = 0 \quad (A-13)
\]

The value of \( P \) that satisfies this equation for neutral stability of a member elastically restrained against rotation at each end is the critical load \( P_{cr} \).

When the restraints at the ends of the column are equal, \( m_a = m_b = m \) and \( \varepsilon_a = \varepsilon_b = \varepsilon \). For this case equation \((A-13)\) reduces to
\[
\left(1 + \frac{G}{3} \varepsilon\right)^2 - \left(\frac{F}{6} \varepsilon\right)^2 = 0 \quad (A-14)
\]

This equation cannot be solved directly for \( P_{cr} \), the critical value of the load \( P \), because \( P \) occurs in
transcendental expressions $F$ and $G$. If $F$ is regarded as known, however, equation (A-14) yields as solutions for $\varepsilon$ two values

$$\varepsilon = \frac{1}{F - G} \quad (A-15)$$

$$\varepsilon = \frac{-1}{F + G} \quad (A-16)$$

which describe two mathematically possible buckling configurations. The buckled shape that corresponds to each of these equations may be obtained by substituting the value of $\varepsilon$ in either equation (A-9) or equation (A-10). From this substitution of $\varepsilon$ as given by equation (A-15) it is found that $\theta_a = -\theta_b$, which by the aid of equations (A-7) and (A-8) shows that $M_a = M_b$. This type of buckling is represented by the bent member of figure 14(a). Similarly, by substitution of the value of $\varepsilon$ as given by equation (A-16) in either equation (A-9) or (A-10), it is found that $\theta_a = \theta_b$, which, again by the aid of equations (A-7) and (A-8) shows that $M_a = M_b$. This type of buckling is illustrated by figure 14(b).

The type of buckling shown in figure 14(b) will occur at a lower load than the type shown in figure 14(a) because of the longer wave length involved. A column with equal elastic restraints resisting rotation at its ends will therefore adopt the buckling configuration shown in figure 14(b) at a load $P = P_{cr}$ that satisfies equation (A-16).

When the values of $F$ and $G$ as given by equations (A-3) and (A-4) are substituted in equation (A-16), the following relation is obtained:

$$\varepsilon = \frac{-2u}{\tan \frac{2u}{2}} \quad (A-17)$$

From equations (A-5) and (A-6)
By use of the relationship expressed by the Euler column formula,

\[ P_{cr} = \frac{c n^2 \pi I}{L^2} \quad (A-19) \]

equation (A-18) becomes

\[ 2u = \pi \sqrt{c} \quad (A-20) \]

Equation (A-17) may therefore be expressed as

\[ \epsilon = \frac{-\pi \sqrt{c}}{\tan \left( \frac{\pi \sqrt{c}}{2} \right)} \quad (A-21) \]

This expression for the elastic restraint at the ends of a column in terms of the fixity coefficient is shown graphically in figure 15 in order to show that the variation in the fixity coefficient is small when the restraint is large.

If the equations (A-11) or (A-12), which define \( \epsilon \), are solved for \( \pi I/L \) when \( m_a = m_b = m \) and this value is substituted in equation (A-19), the Euler column formula can be written

\[ P_{cr} = \frac{c n^2}{L} \frac{m}{\epsilon} \quad (A-22) \]

Substitution of the value of \( \epsilon \) as given by equation (A-21) in equation (A-22) gives

\[ P_{cr} = -\frac{m \pi \sqrt{c}}{L} \tan \left( \frac{\pi \sqrt{c}}{2} \right) \]

Multiplication of both sides of this equation by \( L/m \) results in the equation
\[
\frac{P_{cr}L}{m} = -\pi \sqrt{c} \tan \left( \frac{\pi}{2} \sqrt{c} \right) \quad (A-23)
\]

Equation (A-23) relates the fixity coefficient \( c \) to the critical load, the length of the column, and the magnitude of the elastic restraint. This equation is represented graphically in figure 2.

REFERENCE

## Table 1. Values of $\frac{F_{crL}}{m}$ and Corresponding Values of $c$

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<th>$c$</th>
<th>$\frac{F_{crL}}{m}$</th>
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### TABLE 2

**SUMMARY OF PROPERTIES OF STAINLESS-STEEL COLUMNS**

[All specimens were supplied by Flaxwings, Inc.; series A, B, C, D from roll No. 4400; E from roll No. 2546, heat No. 7129; F from roll No. 659, heat No. 7229; and G and H are type 322.]

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<tr>
<th>Series</th>
<th>Grain</th>
<th>Heat treatment</th>
<th>Hardness description</th>
<th>Ultimate stress (lb/sq in.)</th>
<th>Yield stress (lb/sq in.)</th>
<th>Elongation (percent)</th>
<th>Hardness (Rockwell number)</th>
<th>E (lb/sq in.)</th>
<th>m (lb-in.)</th>
</tr>
</thead>
<tbody>
<tr>
<td>A</td>
<td>Longitudinal</td>
<td>Not heat-treated</td>
<td>Full hard</td>
<td>185,000 to 186,600</td>
<td>149,200 to 149,600</td>
<td>9.5 to 10.0</td>
<td>C-39</td>
<td>25.10 x 10^6</td>
<td>26.3 x 10^6</td>
</tr>
<tr>
<td>B</td>
<td>Longitudinal</td>
<td>400°F for 72 hr</td>
<td>Full hard</td>
<td>---</td>
<td>---</td>
<td>---</td>
<td>---</td>
<td>^25.12</td>
<td>---</td>
</tr>
<tr>
<td>C</td>
<td>Transverse</td>
<td>Not heat-treated</td>
<td>Full hard</td>
<td>165,900 to 168,800</td>
<td>148,200 to 149,600</td>
<td>10.0</td>
<td>C-38</td>
<td>27.35</td>
<td>---</td>
</tr>
<tr>
<td>D</td>
<td>Transverse</td>
<td>900°F for 72 hr</td>
<td>Full hard</td>
<td>---</td>
<td>---</td>
<td>---</td>
<td>---</td>
<td>^28.85</td>
<td>---</td>
</tr>
<tr>
<td>E</td>
<td>Longitudinal</td>
<td>1/2 hard</td>
<td>Full hard</td>
<td>166,600</td>
<td>149,200</td>
<td>26.0</td>
<td>C-34</td>
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</tr>
<tr>
<td>F</td>
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<td>Annealed</td>
<td>Full hard</td>
<td>85,200</td>
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<td>B-68</td>
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</tr>
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<td>G</td>
<td>Transverse</td>
<td>900°F for 1 hr</td>
<td>Full hard</td>
<td>---</td>
<td>---</td>
<td>---</td>
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<td>^29.0</td>
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<tr>
<td>H</td>
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<td>900°F for 1 hr</td>
<td>Full hard</td>
<td>---</td>
<td>---</td>
<td>---</td>
<td>---</td>
<td>29.15</td>
<td>^29.17</td>
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</table>

*These values of E were used to determine m and are the values used in the analysis of the test data.*
### Table 3

Analysis of Test Data Obtained from Series A of Stainless-Steel Column Specimens

#### Evaluation of \( m \)

<table>
<thead>
<tr>
<th>( L ) (in.)</th>
<th>( b ) (in.)</th>
<th>( t ) (in.)</th>
<th>( P_{cr} ) (lb)</th>
<th>( E ) (lb/sq in.)</th>
<th>( I ) (in.(^4))</th>
<th>( c = \frac{P_{cr}L}{m-KI} )</th>
<th>( \frac{P_{cr}}{m} ) (From fig. 2)</th>
<th>( P_{cr}L ) (lb-in.)</th>
<th>( m ) (lb-in.)</th>
</tr>
</thead>
<tbody>
<tr>
<td>2.4505</td>
<td>0.3199</td>
<td>0.02997</td>
<td>118.51</td>
<td>22.86 ( \times 10^6 )</td>
<td>0.71753 ( \times 10^{-6} )</td>
<td>3.8273</td>
<td>0.2690</td>
<td>290.41</td>
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<tr>
<td>2.1882</td>
<td>0.3186</td>
<td>0.03001</td>
<td>145.17</td>
<td>22.85</td>
<td>0.72045</td>
<td>3.7644</td>
<td>0.5741</td>
<td>316.94</td>
<td>562</td>
</tr>
<tr>
<td>1.9110</td>
<td>0.31978</td>
<td>0.0300</td>
<td>187.54</td>
<td>22.85</td>
<td>0.71951</td>
<td>3.7510</td>
<td>0.6546</td>
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<td>548</td>
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</table>

Ave. = 740

#### Evaluation of \( \frac{L}{\rho} \) eff

<table>
<thead>
<tr>
<th>( L ) (in.)</th>
<th>( b ) (in.)</th>
<th>( t ) (in.)</th>
<th>( \rho ) (in.)</th>
<th>( A ) (sq in.)</th>
<th>( P_{cr} ) (lb)</th>
<th>( \frac{P_{cr}}{m} ) (From fig. 2)</th>
<th>( \sqrt{c} )</th>
<th>( \frac{L}{\rho} )</th>
<th>( (\frac{L}{\rho}) ) eff</th>
<th>( \frac{P_{cr}}{A} ) (lb/sq in.)</th>
</tr>
</thead>
<tbody>
<tr>
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<td>0.0300</td>
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<td>0.009589</td>
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<td>0.2580</td>
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<td>1.9716</td>
<td>235.3</td>
<td>143.5</td>
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<td>0.0300</td>
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<td>0.009549</td>
<td>145.17</td>
<td>0.9741</td>
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<td>1.9402</td>
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<td>159.7</td>
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<tr>
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<td>0.009545</td>
<td>527.97</td>
<td>0.7389</td>
<td>3.6975</td>
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<td>0.00858</td>
<td>0.009561</td>
<td>651.84</td>
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<td>0.009496</td>
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<td>0.009552</td>
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<td>0.7030</td>
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<td>1.9264</td>
<td>85.9</td>
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<tr>
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<td>0.6945</td>
<td>3.715</td>
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<tr>
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<td>1009.04</td>
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<td>1.9428</td>
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</table>
Figure 1.— Column and end fixtures assembly.
Figure 2. - Relationship of $P_{cr}L/m$ and $c$ for columns equally elastically restrained against rotation at each end.

$P_{cr}L/m = -\pi \sqrt{c} \tan(\pi/2\sqrt{c})$
Figure 3.- Column curve for full-hard stainless steel, 0.030 inch thick, not heat-treated; loaded with grain. Series A.
Figure 4.-- Column curve for full-hard stainless steel, 0.030 inch thick, heat-treated; loaded with grain. Series B.
Figure 5.- Column curve for full-hard stainless steel, 0.030 inch thick, not heat-treated; loaded across grain. Series C.
Figure 6.- Column curve for full-hard stainless steel, 0.030 inch thick, heat-treated; loaded across grain. Series D.
Figure 7.- Column curve for 1/2-hard stainless steel, 0.030 inch thick, not heat-treated; loaded with grain. Series E.
Figure 8. Column curve for annealed stainless steel, 0.030 inch thick; loaded with grain.
Series F.
Figure 9.—Column curve for full-hard stainless "W" steel, 0.030 inch thick, heat-treated; loaded across grain. Series G.
Figure 10. Column curve for full-hard stainless "W" steel, 0.030 inch thick, heat-treated, loaded with grain. Series H.
Figure 11.—Relative errors in column curves obtained by using different values of $E$ in analyzing series $G$ of the stainless-steel column tests.
Figure 12.—Comparison of derived column curve with Euler curve for different values of $E$. Series G of the stainless-steel column tests.
Figure 13.—Bending of a compressed bar with couples on the ends. (From reference 1.)

(a) $M_a = -M_b$

(b) $M_a = M_b$

Figure 14.—Buckling configurations of a compressed bar with elastically restrained ends.
Figure 15.— Relationship of $\text{mL/MI}$ and $c$ for columns equally elastically restrained against rotation at each end.