SPACE LIFE SUPPORT ENGINEERING PROGRAM

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SUMMARY

This report covers the second twelve months of work performed under a three-year NASA University Grant awarded to Iowa State University to perform research on topics relating to the development of closed-loop long-term life support systems with the initial principal focus on space water management. In the first phase of the program, investigators from chemistry and chemical engineering with demonstrated expertise in systems analysis, thermodynamics, analytical chemistry and instrumentation, performed research and development in two major related areas; the development of low-cost, accurate, and durable sensors for trace chemical and biological species, and the development of unsteady-state simulation packages for use in the development and optimization of control systems for life support systems. In the second year of the program, emphasis was redirected towards concentrating on the development of dynamic simulation techniques and software and on performing a thermodynamic systems analysis, centered on availability or exergy analysis, in an effort to begin optimizing the systems needed for water purification. The third year of the project will be devoted to the completion of the dynamic systems capability, the extension of the thermodynamic analysis to the entire closed loop long term life support system hardware, and the development of a comprehensive model to describe the effect of diet, environment, and exercise patterns on the uptake, redistribution, manufacture, and excretion of water from crew members.
PERSONNEL

Director: R. C. Seagrave, Distinguished Professor of Chemical Engineering

Post Doctoral Research Fellow: Gadiraju Varma (through May 1, 1993)
   Major effort: ASPEN steady-state model development

Graduate Research Assistant: Dasaratha Sridhar (entire period)
   Major effort: Fault Detection Algorithm Development

Graduate Research Assistant: Sharmista Chatterjee (entire period)
   Major effort: Thermodynamic Analysis, ASPEN Model Development
   of Integrated Physical-Chemical and Biological Systems

Graduate Research Assistant: Susan Doty (entire period)
   Major effort: Physiological Water Model Development

Graduate Research Assistant: Anca Stefanescu (since October 1, 1992)
   Major effort: Thermodynamic Analysis and Model Development

Graduate Research Assistant: Megan Scherb (through December 31, 1992)
   Major effort: Thermoregulation Model Development

RESEARCH AND DEVELOPMENT PROGRESS

The present research program is organized in three related, integrated, and high-coupled projects designed to produce usable results within a three-year period. These projects address some of the unique problems of long-term closed-loop life support systems which are designed for periods of greater than 90 days. Among these are:

- Optimization of life-support systems to operate dynamically over a range of conditions which vary with the diet, exercise protocol, and environmental conditions of the crew.
• Optimization of energy consumption and reduction of entropy generation by separation devices, processes, and systems.

• Optimization of process control strategies for conserving energy and material resources while maintaining environmental quality.

Progress during the 1992-93 year has been made in the first two areas, and is summarized as follows.

1. Integration of a working model of the crew into a previously developed system model has been accomplished for the steady state. It is now possible to simulate the behavior of the system at steady state for different crew sizes, of differing physical characteristics and genders, and with different diets. Efforts are continuing during the summer of 1993 to develop a similar material and energy balance model for plants, for incorporation into the overall model. (Results of the crew model are contained in the M. S. thesis of Chatterjee (Appendix 1), while the plant work is being done by an undergraduate summer research student (Tenawati Tanu) who is being co-sponsored by the NASA grant and the Iowa State University Women in Science and Engineering Summer Intern Program.

2. Progress on developing a dynamic model of the crew is occurring in two areas. Susan Doty is developing a dynamic model which will describe the unsteady state distribution of water in the body; that is the uptake, creation, sequestration, and outflow of water in humans as a function of diet, exercise protocol, and environmental factors. Initial development is being done using the dynamic simulation software package EXTEND. Details of this work and some initial results are shown in Appendix 2. The second area of dynamic modeling is the further development of a model of thermoregulatory and fuel depletion aspects of humans-the description of the onset of hyperthermia, hypothermia, dehydration, or glycogen depletion as a function of exercise protocol and environmental parameters. This work, which was performed by Megan Scherb in 1992, is being extended by a summer research student (James Smith) who is also supported by the NASA grant. The results obtained by Scherb are shown in Appendix 3. Both of these efforts will be integrated into the overall ASPEN-based systems model.

3. The importance of availability analysis (exergy analysis) has continued to be a concern as the life support system becomes more complex and as biological interfaces
become more numerous. Our preliminary results in this area are described in a paper by Chatterjee and Seagrave that was presented at the 1993 International Conference on Environmental Sciences in July 1993. A copy is attached as Appendix 4. Present efforts by Anca Stefanescu and Chatterjee are directed towards the development of more rigorous energy and entropy balance equations or unsteady-state, open, complex systems with large numbers of chemical reactions occurring. No such balances are presently available.

RELATED WORK

During the summer of 1993, Research Assistants Dasaratha Sridhar and Sharmista Chatterjee are engaged in research in the Life Support Division at the NASA Ames Research Center, to gain needed practical experience and to acquire background training for the next stages of the project. Sridhar, whose eventual assignment will be to develop algorithms for fault detection in automatically controlled closed loop systems, is working with Mark Ballin doing high-level programming and model development for the process control problem. Chatterjee is working with John Fisher on super-critical water oxidation, a prime candidate technology for the water regeneration loop in the life support system.

APPENDIX


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1 INTRODUCTION

As humans move out from the earth into the Solar system, their most immediate challenge is to provide a life supporting environment in locations that are most devoid of food, air and water. In earlier space flights, life support systems were limited to what may be referred to as open loop configurations. In these configurations, consumables such as water, oxygen and food were stored on board the space craft before the mission. During the mission, these resources were used as necessary and any waste was vented overboard or returned to storage for the duration of the mission. Nothing was recycled.

These open loop ECLSS (Environmental Control Life Support System) configurations worked well for short duration missions. But for long duration missions or to establish a permanent station it is imperative to include regenerative systems in order to reduce the weight of expendables supplied through logistic support.

Autonomy from Earth requires the development of a closed loop ECLSS comprised of human, plant and animal modules. The modules must be integrated with a Waste Management System (WMS) to achieve partial and ultimately long-term self-sufficiency.

Goals and Methods of Approach

Our first goal in this project was to perform a systems analysis of a closed loop ECLSS. This pertains to the development of a model of an existing real system from which to assess the state or performance of the existing system. As far as extraterrestrial life support systems are concerned there are no real systems from
which to derive a model. Therefore, systems analysis is applied to conceptual models obtained from a system design effort. This can serve several purposes:

(1) To quantify approximate reservoir and flux values of the critical components within the system such as oxygen, hydrogen, nitrogen and carbon dioxide.

(2) To compare technology candidates for a technology trade study.

(3) To determine the sensitivity of the system to uncertainty in system variables.

(4) To serve as a basis for future dynamic modelling of the system.

For our modelling purposes we used a simulator tool called ASPEN (Advanced System For Process Engineering). The reasons for choosing ASPEN will be elucidated later on in this thesis.

Our second goal was to evaluate the thermodynamic efficiency of the different components comprising an ECLSS. Use is made of the second law of thermodynamics to determine the amount of irreversibility or exergy loss of each component. This will aid design scientists in selecting the components generating the least entropy, as our penultimate goal is to keep the entropy generation of the whole system at a minimum.
Overview of Life Support System

Under the new space exploration initiative adopted by NASA a permanently manned orbiting platform called Space Station Freedom (SSF) is to be constructed. The most immediate challenge facing scientists is to provide a life supporting environment on the SSF. The open-loop life support systems currently in use by the United States require that the air, food and water needed by humans in space be supplied from the Earth. These same systems are unable to recycle the waste products that result and these must be returned to the Earth or left in space. Since humans require approximately 6.5 kg of potable water, air and food each day, provisions for a crew of four run to nearly 9500 kg per year. In addition, if the crew is to bathe, wash dishes, flush toilets and clean clothes, then the crews needs swells to nearly 28,500 kg of provisions each year, (Rummel, 1990). This is the overboard on space operations. As long as mass to orbit is a quantity to be conserved, then recycling of life support commodities represents one of the best ways to lower this overhead.

The objective of a regenerative ECLSS is to recycle the resources required for the mission. This increases the self-sufficiency of the system and creates what is commonly termed as the Closed Loop Environmental Control Life Support System.

The usefulness of the closed loop system is further corroborated by loop closure studies, (Humphries et al., 1986). They considered the on-orbit and resupply weight and volume of a series of impacts ranging from a completely open loop to a closed
loop water and oxygen recovery system. As illustrated in Figure 1, there is a considerable reduction in cost for a closed loop system.

Figure 1: Loop closure cost summaries

where

OPTION 1 is a completely open system using oxygen resupply, LiOH carbon dioxide removal and water resupply.

OPTION 2 replaces the LIOH system with regenerable carbon dioxide removal.

OPTION 3 added an oxygen generation system and processed the condensate water for reuse in the oxygen generator. Excess hydrogen was compressed and returned to the Earth.
OPTION 4  added hygiene water recovery.
OPTION 5  added urine reclamation to OPTION 4.
OPTION 6  added different types of carbon dioxide reduction.
OPTION 7  combines closed loop oxygen and water recovery systems, adding urine reclamation and potable water recovery to OPTION 6.

Subsystems of the ECLSS

The closed loop life support system can be primarily subdivided into seven subsystems, (Gustavino and Mankamyer, 1989). They are:

(1) THC (Temperature and Humidity Control)
(2) ACS (Atmosphere Control and Supply)
(3) AR (Air Revitalization)
(4) FDS (Fire Detection Subsystem)
(5) WRM (Water Recovery and Management)
(6) WM (Waste Management)
(7) ES (Extra vehicular activity Support)

The functions of the above subsystems are defined in Figure 2.
Figure 2: Functional units of the ECLSS (Adapted from Humphries, et al., 1986)
Types of Life Support Systems

There are two types of Life Support Systems:

- Physico-Chemical type
- Bioregenerative type

The Physico-Chemical type of Life Support is an example of a medium term regenerative design. In this approach the different subsystems of the ECLSS uses physico-chemical methods to recover metabolic oxygen from respiratory carbon dioxide and fresh water from contaminated water. Food and waste are treated as open loop consumables and expendables. However, the supply of food will also constitute a source of water. The primary source of energy is electrical energy.

The Bioregenerative type will provide partial closure of the food loop with food production and partial waste recovery of the plant material (Gustavino and Mankameyer, 1989). The Bioregenerative approach is commonly referred to as the Controlled Ecological Life Support System (CELSS). A CELSS uses plants to imitate the same process that support life on Earth, with one important difference. On Earth there are large buffers to keep the balance between the various elements, but in space these buffers will be minimal. Hence it is very essential to maintain a high degree of control over the process involved. In this system the carbon dioxide which is generated is used by the plants which separate and release oxygen. The carbon is then used along with water and light energy to produce food. After the crop is harvested, the inedible parts of the harvest are used as fertilizers so the portion of the nutrients generated by the plants may be recovered. The primary energy supplier for this design is the sun. Interaction between the different components of CELSS is shown in Figure 3.
Figure 3: Various Components of a Bioregenerative Life Support System
(Adapted from Gustavino and Mankameyer, 1989)

Need For Computer Modeling

In order to achieve reliable and economical life support during the various extra-terrestrial missions, critical and difficult choices have to be made:

- At the system level, such as between storing and regenerating essential supplies.
- At the subsystem level, such as between physico-chemical and bioprocessing for regeneration of oxygen;
• At the compartment level such as between the Bosch or Sabatier process for reduction of carbon dioxide.

Computer modeling can be used to evaluate proposed system designs with respect to the above mentioned criteria without carrying out extensive laboratory experiments. Thus, time and money required for technology testing can be reduced. Computer modeling is also an effective way to check component design specifications and to identify subsystem interface problems. It allows for the examination of critical operating conditions, performance of failure mode analysis and determination of total system performance. This enables the engineer to determine which designs require further development.
Discussion of ECLSS Simulation techniques

In view of the ongoing developments, the applications of an ECLSS simulation tool can be broadly categorised into two divisions, (Simon et al., 1991). These are:

- Mission Analysis
- Assessment of Operational phases.

A major point of interest during mission analysis lies in the budgeting of consumables and disposables such as water and oxygen. For this purpose it is sufficient to determine steady state solutions for most phases of the mission.

For the assessment of operational phases such as the launch, transient modeling of the system is used. Transient modeling is also used in failure mode analysis.

In the area of steady state modeling notable work has been done at the Jet Propulsion Laboratory, California, (Seshan et al., 1991), in developing a Generic Modular Flow Schematic (GMFS) architecture capable of encompassing all functional elements of a physical/chemical life support system. The GMFS is coded in ASPEN. It can be used to compute the material, heat and power flow quantities of every stream in each of the subsystem functional elements for the chosen ECLSS configuration. The input requirements for the GMFS are crew requirements and power data which are obtained from previous bread-board test results. The comprehensive ASPEN material and energy balance output is transferred to a systems and technology assessment spreadsheet for rigorous system analysis and trade studies.

In the field of transient modeling the software to be used by NASA in the 1970's and early 1980's was General Environmental / Thermal and Life Support program (G189A), (Barker et al., 1971). G189A is a complex program written in FORTRAN-5 and FORTRAN-77. It is modularized using numerous subroutines. G189A has a
number of advantages. It offers the advantage of steady-state, quasi-transient or true transient logic. It also has the flexibility of preprocessor and post processor subroutines. However the major limitations of G189A is that it has limited physical property and stream data structures which cannot take into account the different components present in a normal ECLSS. Moreover, advanced technologies can introduce new operations in life support systems which are not modeled by G189A.

In the later half of the 1980's a Computer Aided Systems Engineering and Analysis (CASE/A) ECLSS series was designed as a general ECLSS design and analysis package, (Bangham and Reuter, 1987). This program evolved from both the G189A and SINDA programs and shares the G189A architectural concepts. The designer/analysis interface is graphics based and allows the designer to build a model by constructing a schematic of the system under consideration. CASE/A provides for transient tracking of the flow stream constituents and determination of their state (e.g. temperature, pressure, composition) throughout the ECLSS. However the limitations of G189A are also present in CASE/A.

In the European space arena in 1989 a new software was developed called ECOSIM, which is capable of simulating ECLS subsystems, (Perez and Torroglosa, 1991). ECOSIM provides a graphical language. It is based on a set of icons, each representing a specific ECLSS component the user can select to build a complete system schematic. The ECOSIM simulation is based on a mathematical model of the ECLS system under consideration, its environment and the scenario applied. The basic formalism accepted by ECOSIM is a non-linear, state-space continuous time model subject to discrete events. The solution of the mathematical model is then accomplished using a standard implicit differential algebraic equation solver.
Development of an Integrated Steady State model of a Closed Loop Environmental and Life Support System

Our first goal in this work has been to develop a closed loop steady state physico-chemical process model. The distinctive feature of our model which is not present in any earlier work, (Seshan et al., 1991) is that the input requirements for the various ECLSS subsystems is determined from the user written 'crew' model. A Fortran block is appended to the main program to simulate the basic metabolic operations of the crew. The system is completely closed with respect to free water and oxygen, the only inputs being food which includes dry food (i.e carbohydrates, fat and protein) and water present in food. The utility of the crew model and its salient features will be dwelt upon in the following chapter.

Another feature of our approach is a reservoir concept which was adopted throughout the model to improve its versatality. The outlet flows of some subsystems are not directly connected to the inlet of another, but rather are connected to a reservoir. In addition to reducing the modeling effort, this allows for resource sharing among subsystems and would help in sensitivity analysis of the system.

The model is coded in ASPEN (Advanced System for Process Engineering). ASPEN is an application program developed for steady state, chemical process simulation, (ASPEN PLUS USERS GUIDE, 1988). The program allows the user to model a process flow-sheet by sequencing chemical operations similar to an actual process. The program provides basic 'block' operations, such as pumps, mixers, heat exchangers, compressors and other widely used unit operation blocks that an user can put together to represent a chemical process. Over 1000 chemical species, including gases, liquids and solids are available to the user with physical and
chemical property data included. The program also permits the inclusion of user-written FORTRAN subroutines for custom operational blocks. As mentioned earlier, there are other ECLSS simulators available such as G189A and CASE/A. However, in a study by the NASA Ames Research center to evaluate the existing software for life support modelling (Likens and Force, 1988), ASPEN was rated the highest. Hence ASPEN was chosen for our model. The details of ASPEN modelling of the various ECLSS subsystems will be elucidated in chapter 4.

Of the seven ECLSS subsystems mentioned earlier, our model does not take into account the FDS and the ES. The interactions between the other subsystems with the crew is illustrated in Figure 4.
Figure 4: Relationship between the Crew and the Life Support Subsystems
3 HUMAN HABITAT MODULE

The principal components of the human habitat module are

- Crew
- Cabin where the crew is residing

We shall now elucidate the strategies adopted in modeling the crew and the crew living space.

THE CREW MODEL

Need For Development of the Crew model

Long range goals for the space program include the establishment of a man-tended lunar base and human expedition to Mars as a stepping stone towards expansion of humanity into space (National Commission on Space, 1988, and NASA TM, 1988). The requirements of the human module 'drives' the design and operation of an 'ECLSS'. The outputs from the crew model determine the mass flow rates of the different flow streams into the other subsystems of the ECLSS. The objective here has been to develop a computer model to simulate the metabolic inputs and outputs from the crew module, accounting for the human body mass, age and gender and various categories of physical activities. Metabolic inputs to the human module consists of oxygen, food and water while the metabolic outputs include carbon dioxide, solid feces, urine and insensible water (perspiration and respiration).
Model Development

The crew module primarily consists of two subsystems: inputs and outputs. In developing this model the following assumptions were made:

- With the functional dependence of energy activity upon gravity not being clearly understood due to lack of available experimental data it was assumed that the basic relationships which are obtained for terrestrial data hold under reduced gravity conditions. It is however expected that energy expenditure rate in fractional gravity (e.g. Lunar 1/6 g or Martian 1/3g) is smaller than at 1g for a given activity (Sadeh, 1989).

- The crew is at steady state so there is no accumulation of food or water by any individual, i.e. the weight of the individual remains the same.

The premise of this study is then that all of the input/output streams to the crew module are determined by the energy expenditure rate.

The crew module is now determined based on the following considerations:

- Determination of basal metabolic rate.
- Energy activity
- Metabolic input requirements
- Metabolic wastes.

Basal Metabolic Rate

Basal metabolic rate or basal energy expenditure is defined as the heat expended by an individual at least 10 hours after the last meal, resting in a lying position, awake, at a normal body and ambient temperature and without any physical or physiological stress. The basal metabolic rate is expressed in Kcal per day per square meter of body surface.
The basal metabolic rate is a function of age, body mass and gender. It is evident that there is a non-negligible influence of age on energy expenditure, (Fleisch 1951). Also, it is known that the magnitude of energy expenditure is a function of the amount of active cells in the body or body cell mass. Women have more adipose tissue and less muscle than men, and therefore have a lower metabolic rate than men.

Hence the following equations as suggested by Dreyer in 1920, can be used to compute the basal metabolic energy of men and women from the age of 5 onwards, (Sherman, 1952)

Male: \[ BEE = \frac{W^{0.5}}{0.1015 \times A^{0.1333}} \] (1)

Female: \[ BEE = \frac{W^{0.5}}{0.1127 \times A^{0.1333}} \] (2)

where \( BEE \) : Basal metabolic rate / Basal energy expenditure in Kcal per 24 hours

\( W \) : weight in grams

\( A \) : age in years

**Energy Activity**

The crew members of Space Station Freedom will be engaged in a wide variety of on base "shirt-sleeve" earthlike activities. It is assumed that the crew will be engaged in four basic human physical activities:

(1) Rest/Lying
(2) Walking
(3) Sitting
(4) Running

According to our assumption that terrestrial relationships hold good in the space station, we can use the following relationships to express the different human activities in terms of the basal metabolic rate, (Lusk, 1928).

<table>
<thead>
<tr>
<th>ACTIVITY</th>
<th>ENERGY EXPENDITURE</th>
</tr>
</thead>
<tbody>
<tr>
<td>Lying</td>
<td>BEE</td>
</tr>
<tr>
<td>Walking</td>
<td>2.57BEE</td>
</tr>
<tr>
<td>Sitting</td>
<td>1.23BEE</td>
</tr>
<tr>
<td>Running</td>
<td>7.86BEE</td>
</tr>
</tbody>
</table>

where BEE : basal energy expenditure in Kcal/hr

Figure 5 depicts the influence of the energy activity on the crew.
Figure 5: Metabolic mass flow rates for the human module (Adapted from Sadeh et al., 1989)

**Metabolic Input Requirements**

Metabolic inputs for the human module consists of food (dry food together with water in food), oxygen and water as shown in Figure 5 above. The mass flow rates of these metabolic inputs was expressed in terms of the energy expenditure by the crew in one hour.

**Dry Food Consumption**

Dry food is the main source of energy to the body and consists of macro and micro nutrients and vitamins. The major components of dry food are the macro nutrients which are composed of carbohydrates, proteins and fats. Consequently, for our modelling study here we only have to take the major macro nutrients into consideration.
The normal diet of a person consists is taken as 200g of carbohydrates, 70g of protein and 60g of fat, (Lehninger, 1977). Using the same composition as prescribed by Lehninger for the crew, we can express the mass fraction of protein and fat in the diet in terms of carbohydrate present as

\[ P = 0.35 \times C \]  \hspace{1cm} (3)

\[ F = 0.30 \times C \]  \hspace{1cm} (4)

where

- \( P \) = amount of protein in diet in grams
- \( F \) = amount of fat in diet in grams
- \( C \) = amount of carbohydrate in diet in grams

For our calculation the representative compounds used as typical carbohydrate, fat and protein were Glucose (C\(_6\)H\(_{12}\)O\(_6\)), Palmitic Acid (CH\(_3\) [CH\(_2\)]\(_{14}\) COOH) and Alanine (NH\(_2\)CH [CH] COOH) respectively, (Stokes, 1988).

All the energy required by man is obtained from food. The energy produced through metabolism of one gram of carbohydrate, fat or protein is 4.17, 9.3 or 4.3 Kcal respectively (Bursztein et al., 1989).

Knowing the energy expenditure of a crew member (man/woman) in one hour, the amount of food required is thus computed as

\[ TEE = 4.17 \times C + 9.3 \times F + 4.3 \times P \]  \hspace{1cm} (5)

where

- \( TEE \) = Total energy expenditure

From (3) and (4) the amount of fat and protein can be expressed in terms of equivalent carbohydrates. Thereby, the total carbohydrate required for metabolism is

\[ C = \frac{TEE}{(4.17 + 9.3 \times 0.3 + 4.3 \times 0.35)} \]  \hspace{1cm} (6)
However, it is known that when food is eaten, not all of it is absorbed, some passes through the intestine and is excreted in feces. From literature it is known that approximately only 93% of protein and 98% of fat are metabolized, (Merrill and Watt, 1955, Southgate and Durnin, 1970). On this basis the actual amount of carbohydrate, fat and protein to be supplied in diet can be computed. Thus the total dry food requirement (TDF) is given by

\[
TDF = \frac{C}{0.98} + \frac{F}{0.98} + \frac{P}{0.93}
\]  

(7)

**Oxygen Requirement**

Oxygen is required for the metabolism of carbohydrates, fat and protein. Stoichiometric relations for the metabolism of the three macro-nutrients is given below:

**Carbohydrate**

\[
C_6H_{12}O_6 + 6O_2 \rightarrow 6 CO_2 + 6H_2O
\]  

(8)

**Fat**

\[
C_{16}H_{32}O_2 + 23O_2 \rightarrow 16 CO_2 + 16H_2O
\]  

(9)

**Protein**

\[
2C_3H_7NO_2 + 6 O_2 \rightarrow CH_4N_2O +5H_2O + 5CO_2
\]  

(10)

From the above relations (8), (9) and (10) we can deduce that for metabolizing 1 gram each of carbohydrate, fat and protein we require 0.85, 2.29 and 0.86 liters of oxygen respectively at body temperature and pressure. Thereby

\[
VO_2 = 0.85 \times C + 0.86 \times P + 2.29 \times F
\]  

(11)

where \( VO_2 \) is the amount of \( O_2 \) consumed in liter per hour
Water Requirement

The total water input to the crew consists of metabolic water $H_{2}OM$ (i.e. water produced through the metabolism of food), water in food and free/drinking water. Apart from this the crew also requires water for their daily activities like shower, laundry etc. This is termed as hygiene water.

Metabolic water is produced through the physical and chemical processes involved in the metabolism of dry food. The metabolic water mass flow rate is proportional to the consumption of macronutrients as shown by the stoichiometric relation below. This was derived on the basis of reactions (8), (9) and (10).

$$H_{2}OM = 0.6 \times C + 0.51 \times F + 1.125 \times F$$

The amount of water associated with dry food is 1.32 lb per lb of dry food, (Hall, 1990). Knowing the amount of dry food consumed we can thus evaluate the amount of water entering through food.

Water required for drinking is calculated by performing a water balance over man. Since the man is assumed to be at steady state there is no overall accumulation of water in man. As shown in Figure 5, the output water streams from the crew are urine and insensible water. The mass flow rates of these output water streams is determined as a function of energy expenditure. Hence, water required for drinking is the difference between the total water output and the summation of water input in the form of metabolic water and food water.
Metabolic Wastes

In a closed loop ECLSS the metabolic wastes from the crew are treated, recycled and managed for utilitarian use. Metabolic waste outputs from the crew consists of carbon dioxide, solid feces, urine and insensible water (this includes perspiration and respiration water) as shown in Figure 5. In addition the crew also generates a wide variety of non metabolic wastes such as hygiene water, food packaging wastes and manufacturing wastes. In our model only hygiene water is taken into consideration in this category.

Carbon dioxide

As shown in reactions (8), (9) and (10) carbon dioxide is produced during the metabolism of macro nutrients. Thereby, from the above mentioned relations it is computed that on metabolism 1g of carbohydrate, fat and protein each produces 0.85, 1.59 and 0.71 liters of carbon dioxide respectively at body temperature and pressure.

\[ V_{CO_2} = 0.85 \times C + 0.71 \times P + 1.59 \times F \]  \hspace{1cm} (13)

where \( V_{CO_2} \) is the amount of CO\(_2\) produced in liters in hours.

Feces Production

Here we are only taking into account the solid feces production. An input/output mass balance implies, to a first approximation, that the feces production is equal to the dry food inputs less the amount of food metabolized by the body. Going by the above analogy the total amount of solid feces is

\[ SOLID\ FECES = TOTAL\ FOOD \ - \ METABOLIZED\ FOOD \]
INTAKE
SOLID FECES = \((\frac{C}{0.98} + \frac{F}{0.98} + \frac{P}{0.93}) - (C+F+P)\)

Urine Production

Urine is composed of liquid and solid particulates and daily average values of the various constituents of urine as obtained from (NASA database, 1992) is shown in Table 1.
Table 1: Composition of Urine Analog (Adapted from NASA database)

<table>
<thead>
<tr>
<th>Substance</th>
<th>Amount (person/day)</th>
</tr>
</thead>
<tbody>
<tr>
<td>Water</td>
<td>1454 ml</td>
</tr>
<tr>
<td>Urea</td>
<td>21.0045 g</td>
</tr>
<tr>
<td>Chloride</td>
<td>8.836 g</td>
</tr>
<tr>
<td>Sodium</td>
<td>4.9376 g</td>
</tr>
<tr>
<td>Potassium</td>
<td>3.80746 g</td>
</tr>
<tr>
<td>Sulfate ion</td>
<td>3.251 g</td>
</tr>
<tr>
<td>Creatinine</td>
<td>2.3572 g</td>
</tr>
<tr>
<td>Hippuric acid</td>
<td>1.77958</td>
</tr>
<tr>
<td>Citric acid</td>
<td>0.995362 g</td>
</tr>
<tr>
<td>Glucoronic acid</td>
<td>0.949905 g</td>
</tr>
<tr>
<td>Ammonia</td>
<td>0.849585 g</td>
</tr>
<tr>
<td>Uric acid</td>
<td>0.73359 g</td>
</tr>
<tr>
<td>Tyrosine</td>
<td>0.681862 g</td>
</tr>
</tbody>
</table>

The principal constituent of urine is urea, which is formed during protein metabolism as shown in (10). Thus the amount of urea present in urine is

\[
WRU = 0.332 \times P \tag{15}
\]

where \( WRU \) is the urea produced in grams.
We make an assumption here that the urine urea concentration of the crew remains constant irrespective of the crew exercise level. From Table 1, the amount of water to be associated with the urea to obtain the desired urine concentration is

\[ WIU = WRU \times \frac{1454}{21.0045} \]  

(16)

where \( WIU \) is amount of water in urine in ml.

**Insensible Water Production**

Insensible water is produced by perspiration and respiration from the crew members.

When the crew members are at a resting neutral steady state the body core temperature of an individual remains constant at 98.6 °F. When the crew is engaged in activities which expend energy in excess of basal metabolic activity, the sweat production increases in order to keep the body core temperature at the desired value by removing more heat from the body surface by evaporation of sweat. This is illustrated in Figure 6, which shows sweat production in gm per hour as a function of energy metabolism, (Stegemann, 1977). It is observed that there is a disproportional rise in sweat production in relation to the increase in energy metabolism.

From Figure 6 linear relationships between energy metabolism and sweat production were derived. When energy metabolism is less than or equal to 250 Kcal/hour,

\[ SW = 1.50 \times (\text{energy metabolism}) - 75 \]  

(17)

For energy metabolism greater than 250 Kcal/hour

\[ SW = 2.50 \times (\text{energy metabolism}) - 325 \]  

(18)

where \( SW \) is the amount of sweat produced in grams.
Figure 6: Sweat loss in man at different levels of heat production. (Adapted from Stegeman, 1977)

Water lost by respiration is equivalent to the amount lost in saturating the inspired air. The partial pressure of water vapor in inspired and expired air is 15 and 47 mm Hg respectively, (Seagrave, 1971). Thereby total volume lost as respiration is the difference in volume of water in expired and inspired air. Procedure to calculate the total volume of inspired and expired air will be dealt with in the next section.
CABIN MODEL

The cabin model is used to maintain the requisite environmental criterion in the cabin at 20.8% oxygen, 76.9% nitrogen, 0.3% carbon dioxide and 73.7% relative humidity at 70 °F (Humphries et al., 1986). In order to maintain the cabin at this desired level of habitability we need to regulate the amount of air flowing into the cabin.

Within the cabin there is continuous gas exchange between the cabin air and the crew. The crew is continuously breathing in oxygen rich air and breathing out carbon dioxide rich air.

Hence prior to determining the air flow rate into the cabin it is imperative to calculate the mass flow rates and composition of the inspired and expired air. We propose to evaluate the mass flow rates and composition of the inspired and expired air based on the compartmental model of the human respiratory tract. The respiratory system of a human being includes the nose, nasal cavity and sinuses, the pharynx, the larynx, the trachea and smaller conducting passageways leading to the exchange surfaces of the lungs, (Martini, 1989).

The inspired air which enters through the nose is filtered, humidified and warmed in the respiratory tract before reaching the alveoli in the lungs where gaseous exchange takes place. We shall now outline the procedure for modeling this entire respiratory mechanism.

We start with the compartment model of the human lung, (proposed by Seagrave 1971) to develop a more detailed model of the respiratory tract. It takes into account the volume and heat changes undergone by the inspired air by dividing the respiratory tract into three compartments. These are
(1) Upper respiratory tract where the inspired air is heated to the body temperature of 98.6 °F.

(2) Lower respiratory tract where the inspired air is saturated with water vapor. The partial pressure of water vapor at body temperature is 47mm Hg.

(3) Lungs, where O2 which is required for metabolism is lost from the air to the blood in the pulmonary circuit and CO2 which is produced during metabolism is added back to the air from the blood. Figure 7 depicts the model with all the input and output streams.
Figure 7: Compartmental model of the Respiratory tract.
The basic assumptions in our model are:

1. All the compartments are considered to be homogenous and well mixed so that concentration of each species at any time is uniform throughout each compartment.

2. The total pressure of air at any position within the respiratory tract is 760 mm Hg.

3. Ideal gas law holds in each compartment.

4. The total volume of oxygen required for metabolism does not rise above 3 lit/min at any instant, i.e. the maximum permissible exercise limit of the crew is about 12 times the basal metabolic rate.

In our model we will follow only four significant molecular species in the gas and assume that any other species are present in negligible amounts. The four species to be followed are nitrogen, oxygen, carbon dioxide and water. The fundamental principle which is used to compute the mass flow of each species is

\[
\text{rate of accumulation} = \text{input flow rate} + \text{generation} - \text{degradation or consumption}
\]

Since this is a steady state model the rate of accumulation is equal to zero. The detail of equations used for each compartment is given in Appendix 1.

**Upper Respiratory Tract**

No material exchange is taking place. The inspired air is heated to body temperature. The relation between the volume of air entering and leaving the compartment is given as

\[ Q_1 = 1.054 \times Q_2 \] (19)
where $Q_1$ is the volume of air entering in liters per hour
$Q_2$ is the volume of air leaving in liters per hour.

**Lower Respiratory Tract**

The heated inspired air is saturated with water vapor. The volume of air leaving the compartment can be expressed as

$$Q_3 = 1.045 \times Q_2$$

where $Q_3$ is the volume of air leaving the compartment in liters per hour.

Knowing that the air leaving the compartment has a partial pressure of water of 47 mm Hg and that there is no exchange of other species namely oxygen, carbon dioxide and nitrogen, the mole fractions of the components in the gas exiting the lower respiratory tract is calculated as

- Mole fraction of oxygen ($Y_{O_2}$) = 0.199
- Mole fraction of nitrogen ($Y_{N_2}$) = 0.735
- Mole fraction of carbon dioxide ($Y_{CO_2}$) = 0.004
- Mole fraction of water ($Y_{H_2O}$) = 0.062

**Lungs**

The following material balance equations can be written for the lungs.

- Volume of $O_2$ inspired = Volume of $O_2$ expired + Volume of $O_2$ lost to blood
  $$0.199 Q_3 = Q_4 \times Y_{O_2} + V_{O_2}$$

- Volume of $CO_2$ inspired = Volume of $CO_2$ expired - Volume of $CO_2$ gained from blood
  $$0.004 Q_3 = Q_4 \times Y_{CO_2} - V_{CO_2}$$

- Volume of $N_2$ inspired = Volume of $N_2$ expired
Mole fraction of gases in expired air = 1.0

\[ Y_{CO2} + Y_{N2} + Y_{O2} = 0.938 \]  

(24)

where

- \( Y_{CO2} \) is mole fraction of CO\(_2\) in expired air
- \( Y_{N2} \) is mole fraction of N\(_2\) in expired air
- \( Y_{O2} \) is mole fraction of O\(_2\) in expired air
- \( Q4 \) is volume of expired air in liters per hour

From Figure 8, (Passmore and Robson, 1969) it is found that pulmonary ventilation (VE) increases in proportional to volume of O\(_2\) uptake at STP upto a certain point which is equal to 3 liters per minute of VO\(_2\) beyond which VE increases more than the corresponding O\(_2\) uptake.

**Figure 8:** VE in relation to VO\(_2\) (Adapted from Passmore and Robson, 1969)
In accordance with our third assumption it is justified to assume a linear relationship for $O_2$ uptake and $VE$. This is given by

$$VE = 24.55 \times MO_2 + 0.1125$$  \hspace{1cm} (25)

where $VE$ is volume of gas entering the lungs in liters per minute at body temperature and saturated conditions,

$MO_2$ is volume of $O_2$ required for metabolism in liters per minute at STP and dry conditions.

Knowing the volume of inspired and expired air and their respective mole fractions, we employ the ideal gas relation \((PV = mRT/M)\) to calculate the total mass flow rate of each gaseous component in expired and inspired air.

**ASPEN model of the cabin**

For modeling purpose the cabin is modelled as two compartments in series. The inputs and outputs of each compartments is shown in Figure 9.
Figure 9: Model of Crew Habitat

There is a constant leakage of \(6.5 \times 10^{-4}\) lb/hour of gas from the cabin, (LiSSA, 1992).

For the purpose of maintaining a comfortable living condition within the cabin we need to regulate not only the composition of air within the cabin but also the temperature and air motion. The design flow rate of air into the cabin is calculated from the following relation, (Haines, 1988)

\[
\text{CFM} = \frac{q_s}{T_D \times AF}
\]  
(26)

where

- CFM is design flow rate in cubic feet per minute
- \(q_s\) is sensible cooling load in btuh
- \(T_D\) is design temperature difference between cabin and supply air
- AF is the air factor

The Air Factor is used to take into account the variation in air density. The Air Factor is given by, (Haines, 1988):

\[
AF = \text{Density} \times SH \times 60
\]  
(27)
where Density is air density at design temperature and pressure
SH is specific heat of air, btu per lb, at design temperature
and pressure

For standard conditions, AF is equal to 1.08 btuh/cubic feet per min °F

As considerable energy will be expended in cooling the supply air to a low
temperature we assume that the supply air is nearly at the room temperature of
69°F.

The cooling load is equal to the amount of heat lost by the crew. Since the crew is
at a steady temperature the total heat lost by the crew is equal to their metabolic
activity.

The ratio of CFM and inspired air is then used as a basis to preset the ratio
between INSPIRED AIR and AIROUT.

In the ASPEN model the first compartment (CAB1) was modelled as a MIXER
and the next compartment (CAB2) was modelled as FSPLIT with the volume ratio
between INSPIRED AIR and AIROUT being predetermined. A feedback control
mechanism was used to regulate the amount of air flowing into the cabin. The mass
flow rates of the different gaseous components in the inspired air were the set
variables, while the corresponding mass flowrates of the air into the cabin i.e AIRIN
were the manipulated variables. This was accomplished in our ASPEN model using
Design Specifications.
This chapter discusses different functional units comprising each subsystem of the closed loop ECLSS. Though each subsystem can have a wide variety of technologies, for our modeling purpose either one or two sample technologies were chosen for each subsystem. The modeling technique adopted was either to simulate each major piece of equipment or to simulate the functional performance of the functional unit.

For our modeling purpose we used ASPEN. The reasons for choosing ASPEN over other chemical process simulators, were elucidated in chapter 2.

However certain difficulties were encountered in modeling with ASPEN. ASPEN is not equipped to provide detailed models of electrolyte process, so the electrolysis model, as in water electrolysis process, lacks fidelity in kinetic characterization of electrode reactions. No provision is made in ASPEN for the participation of electrons in the electrode reactions so the cell reactions are modelled as chemical rather than electrochemical process.

Another difficulty is encountered in modeling membrane technologies. ASPEN has no routines for modeling membrane based separations. The membrane based CO2 removal system was therefore modeled using a user written FORTRAN block.

Lastly, ASPEN is a steady state modeling tool and so difficulties are to be encountered in modeling dynamic process.

**Water Reclamation Subsystem**

Water reclamation involves process to reclaim water from sources such as carbon dioxide reduction, cabin humidity condensates, wash water (from clothes, dishes
and hygiene) and urine. Usually the average flow rates of wash water and urinal flush water are taken as 12 lb/day and 1.09 lb/day respectively, (NASA database, 1992).

The flow rate of urine is determined from the metabolic rate of the crew (shown in chapter 3). The procedure to calculate the amount of humidity condensate will be outlined in the section on Atmosphere Humidity Control.

The basic functions of the water reclamation system can be listed as, (Huttenbach and Radford, 1965):

• Collection of waste water.
• Pre-treatment of waste water to inhibit the growth of bacteria and improve the performance of phase change process.
• Reclamation of water using a multifiltration, membrane or phase change process. In general membrane and multifiltration processes are used to treat less contaminated streams such as humidity condensate and CO2 reduction process as while phase change processes are used to recover water from urine and hygiene water.
• Post treatment of recovered water to remove residual contaminants.
• Storage, distribution and thermal conditioning of recovered water.
• Monitoring the quality of recovered water.

Multifiltration

A multifiltration process includes a particulate/bacterial filter, an activated charcoal canister, an anion exchange resin bed and a cation exchange resin bed, (Popoma and Collins, 1965).
Membrane Process

Membrane Process include Reverse Osmosis and Electrodialysis. Generally spiral wound, tubular and tube side hollow fiber membranes are used. The membrane processes have the advantage of less power consumption and heat rejection over conventional phase change process. However, they have the disadvantage that the water to be treated by these processes require a greater degree of pretreatment as compared to conventional phase change process. Moreover, most conventional membranes are unable to withstand high temperatures sufficient for sterilization (Schubert, 1983).

Phase Change Process

Phase change processes involve distillation and condensation, with a goal to retain the solutes (in sterilized form) in the evaporator and to reclaim the energy required for the vaporization process. The phase change processes that are commonly used are air evaporation, vapor compression distillation, thermo electric integrated membrane evaporation subsystem, and vapor phase catalytic amonia removal.

Since phase change processes are used for the treatment of bulk waste water such as urine and hygiene water, we have only considered the phase change process for our water regeneration process. Two sample technologies which were studied are Vapor Compression Distillation (VCD) and Thermo electric Integrated Membrane Evaporation Subsystem (TIMES).

Vapor Compression Distillation

Process Description:

A schematic of a VCD is shown in Figure 10. The recovery of latent heat in the VCD is accomplished by compressing the vapor to raise its saturation temperature and
then condensing the vapor on a surface which is in direct contact with the evaporator. The resultant heat flux from the condenser to the evaporator is sufficient to evaporate an equal mass of water. Thus the latent heat of condensation is recovered by the evaporation process and the only energy required for the process is that necessary to compress the vapor and to overcome the thermal and mechanical inefficiencies. The VCD process occurs in a 70°F to 95°F temperature range by maintaining a nominal condenser pressure of 0.7 psia, (Quattrone, 1981).

As shown in Figure 10, there are two tanks, the recycle tank and the waste water tank which are maintained at a pressure of 20 psia. The still is evacuated to a saturation pressure of 0.5 psia operating at a temperature of 70 °F. Pretreated urine and waste water are stored in the waste water tank and is then sent to the recycle tank. The process is continued till the solids concentration in the recycle loop increases to greater than 50%. The process is then stopped, the brine is removed and the VCD is then restarted.
Figure 10: Schematic of VCD (Adapted from design proposed by Lockheed INC)

The ASPEN model of the VCD is shown in Figure 11. The evaporator and the condenser are modeled as a combined unit using a SEP block. This is valid because there is no heat exchange from the environment for the combined unit. Since ASPEN can only model a steady state process, we have assumed that a 50% concentrated brine is obtained from the evaporator. This eliminates the presence of a recycle loop. This model predicts a 99.8% recovery of waste water which is greater than the literature values of 96%, (Schubert, 1983).
Figure 11: ASPEN model of VCD

Thermo electric Integrated Membrane Evaporator Subsystem (TIMES)

Process Description (Winkler, 1982):
The TIMES is a vacuum distillation subsystem that combines a hollow fiber membrane evaporator with a thermo electric heat pump.

The waste water is first pretreated with sulfuric acid/chromium trioxide solution to provide bacteria control. It is then heated to near the pasteurization temperature of 150 °F by a heat exchanger in contact with the hot junction surfaces of a thermo electric heat pump. The heated waste water then passes through a tubular hollow fiber membrane evaporator. The outside surface of the membrane tubes are exposed to a low pressure of 2.47 psia. Water diffuses through the tube walls and evaporates, forming steam at the tube outer surface while other liquid and dissolved solid constituents of the waste water remain in the recycle loop. The slightly cooled and concentrated waste water is then returned to the recycle tank. The solids concentration in the recycle loop gradually increases until a solids concentration of around 40% is reached. At this point the process is stopped, and the recycle tank and the filter are removed using maintenance probes. Fresh recycle tank and filter units are installed and the process is continued.

Evaporated steam is condensed to water on a chilled porous plate surface that thermally contacts the cold junction surface of the thermo electric heat pump. The condensate is then drawn into an accumulator. Figure 12 shows the schematic of the process.
The ASPEN model of the process is shown in Figure 13. Our approach here is to model the nature of the operation of TIMES, not the actual unit operation blocks involved. The thermo electric junction is modeled using two heaters, one for the hot junction and the other for the cold junction with heat being transferred from the hot to the cold junction. Since ASPEN has no tailor made blocks to model membrane separation we use a SEP block to simulate the operation of the membrane separator. The steam leaving the evaporator is saturated at the evaporator temperature. From the steam tables the saturation pressure of the steam is 3.784 psia. For the purpose
of developing a steady state model we assume that the brine from the evaporator is 40% concentrated with solids and that there are no recycle loops. Predicted recovery of waste water from our simulation calculation is 99.7%.

Figure 13: ASPEN model of TIMES

**Air Revitalization System**

The air revitalization system (ARS) is a critical component of the life support structure on a spacecraft or a planetary base. The ARS controls the temperature, pressure and composition of the cabin. The subsystems included in the ARS are Trace Contaminant Removal, Humidity Control, CO2 removal and reduction and N2 and O2 production, storage and distribution. Different modules are available for each subsystem. We shall now give a brief outline of the process schematic of each module followed by its ASPEN model.
Trace Contaminant Removal Subsystem

Contaminants in a manned space craft emanate from both the crew and the equipment. As mission durations, vehicle sizes, crew sizes, vehicle payload and experimental complexities increase and as the space craft leak rates decrease there will be a concomitant increase in the concentration of the trace stream. For our modeling purpose the flow rate of the contaminant stream was multiplied by the size of the crew. The following technologies are available for contaminant removal, (Ammann, 1989):

- condensing heat exchanger
- freeze out
- decomposition by corona discharge
- adsorption on molecular seives
- catalytic oxidation
- chemisorption

Table 2 shows the generation rates of the different trace stream components. This is based on a basis of 1 crew member.
Table 2  Composition of Trace Contaminant Stream (Adapted from Ammann, 1989)

<table>
<thead>
<tr>
<th>Component</th>
<th>G [ mg/day ]</th>
</tr>
</thead>
<tbody>
<tr>
<td>CO</td>
<td>3000</td>
</tr>
<tr>
<td>CH₄</td>
<td>300</td>
</tr>
<tr>
<td>C₂H₆</td>
<td>2</td>
</tr>
<tr>
<td>C₂H₄</td>
<td>0.05</td>
</tr>
<tr>
<td>C₂H₂</td>
<td>0.5</td>
</tr>
<tr>
<td>H₂</td>
<td>260</td>
</tr>
</tbody>
</table>

For our study the catalytic oxidation process was chosen.

**Catalytic Oxidation Process**

Process Description:

Figure 14 shows a schematic of the process. The process air initially passes through a pre filter which removes the harmful substances present like ammonia or freon which act as catalyst poisons. The bulk of air then flows into the LTCO (Low Temperature Catalytic Oxidizer) which operates at ambient temperature. The CO is oxidized in the LTCO. A small portion of the air which is adjustable, depending on the CH₄ concentration (i.e., higher the CH₄ concentration the greater the flow rate) flows into the HTCO (High Temperature Catalytic Oxidizer). Prior to entering the HTCO, the bypass flow passes through a heat exchanger (HX) and an electrical heater (EH). The temperature to be maintained in the HTCO is between 400 - 450°C. In the HTCO all the higher hydrocarbons present in the trace gas are oxidized. The
recombined flow from the HTCO and the LTCO leaves via a post filter, the purpose of which is to adsorb the acid decomposition products which are generated by catalyst contact and the outlet particle filter.

Figure 14: Schematic of the Catalytic Oxidizer Process (Adapted from Ammann, 1989)
The ASPEN model of the Catalytic Oxidation technology is shown in Figure 15. A user written Fortran block is used to determine the split fraction of gas going into the HTCO. The split fraction depends on the CH₄ concentration in the feed and varies from 0.075 to 0.3. The two reactors were modeled using RGIBBS, which calculates reaction equilibrium based on Gibbs energy minimization.
Atmosphere Humidity Control

The temperature and humidity of a habitat is to be regulated in order to provide a comfortable working environment. For our model we used a condenser-separator module to control the cabin humidity level. Our goal is to cool the gas from the trace removal subsystem to a temperature such that amount of water vapor in the exiting gas is equal to the amount of water vapor which is to be supplied to the cabin to maintain the cabin humidity level. The condensate is treated to obtain water of potable quality. The absolute humidity (mass of water/mass of dry air) is thus known.
Determination of the temperature of the condenser involves an iterative technique, which involves reading enthalpy values of the gas and liquid water from psychometric charts.

This technique cannot be easily adopted for modeling purpose. For our modeling purpose we assume that the gas leaving the condenser is saturated with its moisture content. The temperature of the exiting gas is given by the following relation, (Treybal, 1988):

\[ t_{G2} = t_{G1} - \frac{\lambda}{C_s} (Y_{in} - Y_{out}) \]  (28)

where:
- \( t_{G2} \) = temperature of outgoing gas from condenser
- \( t_{G1} \) = temperature of incoming gas to condenser
- \( \lambda \) = latent heat of vaporization 1075.8 Btu/lb
- \( C_s \) = humid heat
  \[ = 0.24 + 0.45 Y_{in} \text{ Btu/l( lb of air).°F} \]
- \( Y_{in} \) = absolute humidity of gas in
- \( Y_{out} \) = absolute humidity of gas out

For the ASPEN model the condenser and the separator is modeled as a single unit using a SEP block. Knowing the absolute humidity of outgoing air the split fraction of the SEP block is determined in a user written FORTRAN block. Likewise, the temperature of the outgoing air is also determined using relation (28). A provision has however been provided for in the FORTRAN block that if the required temperature of the condenser falls below 40°F, the condenser temperature is to be set to 40°F. This option is provided to conserve energy.
According to the NASA requirement, in order to maintain comfortable living conditions in the crewed habitat the partial pressure of CO2 must not exceed 3 mm Hg, (Ray et al., 1987). In order to achieve this requirement the excess CO2 in the air (generated due to food metabolism and in the trace removal subsystem) is removed in the CO2 removal subsystem. The CO2 that is removed is concentrated and then sent to the CO2 reduction subsystem.

The commonly used CO2 removal subsystems are:

- Solid-Amine Water desorbed CO2 concentrator
- Electrochemical depolarized CO2 concentrator
- 2 or 4 - Bed Molecular sieve
- Metal oxide process where the CO2 is complexed with a solid metal oxide
- Liquid - Sorbent / Membrane - Contact system
For our study the Liquid - Sorbent / Membrane - Contactor system was chosen. The reasons for choosing this system are:

- light weight
- requires no consumables or expendables
- simple and does not rely on other ECLSS subsystems

Process Description (Mc Cray et al., 1990):

CO₂ rich air is circulated through one side of the hollow-fiber membrane contactor and CO₂ is absorbed into the liquid sorbent, which is circulated on the other side of the membrane contactor. The CO₂ rich liquid sorbent is then sent to the second hollow fiber membrane contactor, where the CO₂ is removed using a vacuum compressor. The regenerated sorbent is then recycled to the absorption membrane contactor. A schematic of this process is shown in Figure 17.

Figure 17: Schematic of Membrane based CO₂ removal system (Adapted from Ray et al., 1990)

There being no tailor made blocks for membrane separators in ASPEN, our goal was to model the process objective rather than the actual unit operation. This was
accomplished by the use of a SEP block. The SEP block removes the CO₂ from the air which is in excess of the amount to be present in the cabin air in order to satisfy the cabin atmosphere requirements of 0.3% CO₂. The separation degree of the SEP block is regulated by a user incorporated Fortran block. The air is cooled to 69°F before being sent to the cabin. The CO₂ is sent to the CO₂ reduction subsystem.

Carbon dioxide Reduction Subsystem

Among several technologies known for reducing CO₂, the Sabatier and Bosch process are the two primary options being considered for application in a regenerable ECLSS. Both process reduce CO₂ by catalytic reaction with hydrogen. The Bosch process produces carbon and water while the Sabatier process produces methane and water. Following is a brief discussion of each of these processes.

Bosch CO₂ reduction

Detailed description of the Bosch process, its operation, specific hardware and performance are discussed extensively in the literature (Holmes, King and Keller, 1973, King and Holmes, 1976, Schubert, 1974, Manning and Reid, 1975). We shall give a brief summary of the process followed by its ASPEN model.

Process Description:

The Bosch reaction occurs at 800 to 1000K (980 to 1340 °F) in the presence of various catalyst such as iron, cobalt, nickel or ruthenium/iron alloys (Manning, Reid and Sophonpanich, 1982, Kolodney, Lange and Edeen, 1991). CO₂ combines with H₂ and produces carbon and water vapor as indicated in the overall reaction:

\[
\text{CO}_2 + 2\text{H}_2 \rightarrow \text{C} + 2\text{H}_2\text{O} + 2280 \text{ kJ/kg CO}_2 \quad (29)
\]

The reaction mechanism however involves several reactions:

\[(A) \quad \text{CO}_2(g) + \text{H}_2(g) \leftrightarrow \text{CO}(g) + \text{H}_2\text{O} (g)\]
(B) \[ \text{CO(g) + H}_2\text{(g)} \leftrightarrow \text{C(s) + H}_2\text{O(g)} \]

(C) \[ 2\text{CO(g)} \leftrightarrow \text{C(s) + CO}_2\text{(g)} \]

(D) \[ \text{C(s) + 2H}_2\text{(g)} \leftrightarrow \text{CH}_4\text{(g)} \]

Reaction A occurs first, the others can then occur in parallel. Reaction (C) may not be significant but reaction D is, (Minemoto et al., 1989). However it has been shown that if two reactors are used in series one operating at a higher temperature than the other, the CH4 concentration is negligible in all streams, (Minemoto et al., 1989). In practice, single pass efficiency through the bosch reactor is less than 10% for iron catalysts. Complete conversion is attained by recycling the processed gases with continuous deposition of carbon and removal of water vapor. Since all the reactants entering the subsystem are eventually converted to carbon and water, the overall stoichiometric ratio of two moles of H2 to one mole of CO2 must be maintained, (Spina and Lee, 1985). The schematic of the process is shown in Figure 18.

Figure 18: Schematic of Bosch process (Adapted from Minemoto et al., 1989)
The high purity CO2 recovered by the CO2 removal process and H2 for the reduction are mixed, and this gas is taken to a buffer tank. Then the circulating fans channels the gas through the first and second reactors. The first reactor is maintained at a temperature of 1300 [K] while the other reactor is maintained at 900 [K]. In the first reactor reaction (A) occurs while reaction (B) is predominant in the second reactor. Methanation occurs to an appreciable limit in the second reactor. To prevent the accumulation of CH4 within the subsystem which would eventually lead to a shut down of the process there is a bleed line connected to the trace removal subsystem. The solid carbon is collected in the second reactor, water vapor generated is condensed in the condenser and the unreacted gas is returned again to the buffer tank to be recycled again.

The reactors of the bosch system were modeled using the RGIBBS equilibrium reactor model. The validity in using RGIBBS is checked by estimating the conversion for a single pass (i.e. no recycle). From the ASPEN model net conversion is found to be 37.6%. This is so because two reactors are used in series. Moreover in actual reactors 'graphite equilibrium' is usually not attained due to insufficient residence times and catalyst deactivation. Similar results were reported in literature, (Sacco and Reid,1978). In our model, the H2 feed rate is determined in a user written Fortran block to ensure CO2/H2 stoichiometric ratio is satisfied. Figure 19 shows the ASPEN model of the Bosch system.
Figure 19: ASPEN model of Bosch system
Sabatier Reactor

Process Description:

The Sabatier reaction occurs at 350 to 980 °F in the presence of catalyst. The reaction mechanism is shown below

\[
\text{CO}_2 + 4\text{H}_2 \rightarrow \text{CH}_4 + 2\text{H}_2\text{O} + 1790 \text{ btu/lb CO}_2 \tag{30}
\]

With a stoichiometric reactant mixture the reaction efficiencies are higher than 98%, (Spina and Lee, 1985). So a single pass is always used for a Sabatier reactor. The methane and water vapor formed along with the excess reactants are cooled in the condenser where the water is condensed. The remaining methane and other gases are directed to the trace removal subsystem. Figure 20 shows the system schematic.

![Figure 20: Functional schematic of the Sabatier (Adapted from Spina and Lee, 1985)](image)

The ASPEN model of Sabatier is shown in Figure 21. The reactor is modeled using a RSTOIC. The reason for choosing RSTOIC is that since the conversion per
pass in the Sabatier reactor is usually higher than 98%, a RSTOIC with a specified conversion of carbon dioxide of 98% can adequately simulate the real process. A user written Fortran block is incorporated to direct the stoichiometric amount of H₂ to the reactor.

Figure 21: ASPEN model of the Sabatier

Oxygen generation subsystem

To meet the needs of the crew on board the space station O₂ and H₂ are to be supplied by an O₂ generation assembly. Apart from metabolic consumption, O₂ is required to make up for cabin leakage as well as air lock losses. H₂ is required for
CO2 reduction. For our study we have chosen the Static Feed Water Electrolysis system.

Electrolysis Process:

In a water electrolysis cell, water is dissociated into its elements by supplying electrons to a negatively charged electrode (cathode) to produce H2 and removing electrons from a positively charged electrode (anode) to produce O2. The half-cell reactions for the water electrolysis cells using an alkaline electrolyte are shown below: (Fortunato et al., 1988)

At the cathode:

$$4H_2O + 4e^- \rightarrow 2H_2 + 4 OH^-$$

At the anode:

$$4 OH^- \rightarrow O_2 + 2H_2O + 4e^-$$

Overall reaction:

$$\text{electrical energy} + 2H_2O \rightarrow 2H_2 + O_2 + \text{heat} \quad (31)$$

**Static Feed Water Electrolysis**

Process Description:

Initially the feed water compartment is filled with water and the electrolyte retention matrix is fully charged with aqueous electrolyte. As electrical power is supplied to the electrodes, water is electrolyzed from the electrolyte retention matrix. This increases the initial electrolyte concentration gradient between the water in the feed water compartment and the electrolyte in the electrolyte retention matrix. Water vapor then diffuses from the water feed membrane into the electrolyte retention matrix due to this gradient. Advantages of this system are:

- since there is no contact between the liquid feed water and the cell electrodes ultrapure water need not be used.
• the water feed membrane is liquid impermeable but water vapor and gas permeable eliminates any concerns regarding H2 diffusion into the water cavity.
• no dynamic phase pump/separators are needed to remove the product gas from the feed water stream.

Table 3 shows the design requirements for the process.

Table 3  Performance Requirements of Static Feed Water Electrolysis unit
(Adapted from Fortunato et al., 1988)

<table>
<thead>
<tr>
<th>Parameter</th>
<th>Units</th>
<th>Operational mode</th>
</tr>
</thead>
<tbody>
<tr>
<td>/O2 partial pressure</td>
<td>psia</td>
<td>2.83 - 3.35</td>
</tr>
<tr>
<td>O2 delivery pressure</td>
<td>psia</td>
<td>20±5</td>
</tr>
<tr>
<td>H2 delivery pressure</td>
<td>psia</td>
<td>25±5</td>
</tr>
<tr>
<td>H2O supply temperature</td>
<td>F</td>
<td>65 - 80</td>
</tr>
<tr>
<td>H2O supply pressure</td>
<td>psia</td>
<td>30 - 45</td>
</tr>
</tbody>
</table>

Schematic of the Static Feed Water Electrolysis process is shown in Figure 22.
Figure 22: Schematic of the Static Feed water Electrolysis process (Adapted from Fortunato et al., 1988)
For the ASPEN model of the Static Feed Electrolysis, the water from the water regeneration subsystem is pressurized to 30 psia before being fed to the water electrolyzer. A cooler placed before the electrolyzer removes the heat generated due to compression in order to supply water to the cell at the ambient temperature. Since ASPEN does not have any blocks to simulate the electrolysis process the overall process was modeled as a chemical reaction using a RSTOIC block with the reaction stoichiometry specified and the conversion of water set at 100%. The output from the electrolyzer is then sent to a SEP block which is a combination of a cooler, a depressuiser and a condenser. H₂ and O₂ are sent to their respective buffers. The ASPEN model of this process is shown in Figure 23.
Figure 23: ASPEN model of Static feed water electrolysis
Solid Waste Management

Waste management is required to control the accumulation of materials that would otherwise lead to breakdown of the life support system. Apart from solid wastes generated by the crew, the other physico-chemical processes which constitute the life support system generate typical solid wastes which include particle filters, product carbon cartridges, depleted catalysts, contaminated adsorbents, waste filters, wicking units, chemical and multifiltration units. Various technologies which can be used for solid waste treatment range from use of chemical germicides to stabilize fecal bacterial gas production to complete oxidation of organic wastes, (Quattrone, 1981). Again there are two types of oxidation: wet oxidation and dry oxidation (incineration). For our modeling purpose we used the wet oxidation process.

Wet Oxidation Process

Process Schematic, (Takahashi and Ohya, 1985):

Wet oxidation is a reaction in which organic or reductive compounds are oxidized in the presence of liquid water. This reaction occurs between 100 and 374 °C which is the critical temperature of water. The reaction is carried out in an autoclave to prevent evaporation of water. The pressure is set to 1067 psia. The added O₂ index is maintained at 1, i.e the exact quantity of O₂ is added to completely oxidize the raw materials. There is a linear relationship between % of carbon in waste and the temperature to be maintained in the oxidation process, (from Figure 1 of the paper by Takahashi and Ohya, 1985). The linear relationship is:

\[
\text{% of carbon in feed} = -0.65 \times \text{oxidation temp} + 194
\]  

(31)
For the ASPEN model of the wet oxidation process, the oxidizer was modeled using a RSTOIC block. The temperature was determined from a user written Fortran block using the linear relationship of equation (31). We have assumed that only the carbon content of the waste is completely oxidized to CO₂ while the nitrogen and the hydrogen components of the feed remain unchanged. Thus the ECLSS loop is not completely closed as far as nitrogen and hydrogen are concerned, and a buffer must be used.

![Figure 24: ASPEN model of the wet oxidation process](image-url)
In order to compare quantitatively the merits and demerits of storage vs regeneration of the critical constituents of a life support system (LSS) a detailed systems analysis is required. Herein, we present the results which were computed from the ASPEN model of the ECLSS, which give us a perspective of the requirements and design of the ECLSS. The results can be interpreted to arrive at three categories of information:

- Mass balance sensitivity of the ECLSS.
- Technology trade study
- Parameter sensitivity analysis

**Development of the integrated ECLSS model**

As elucidated in Chapter 3, the crew is the primary producer of wastes. These wastes serve as inputs to the different life support modules. Since the objective of a regeneration design is to reduce the amount of expendables supplied, the wastes are processed in different LSS modules before being recycled back.

The crew specification will then influence the mass flow rates of the different components within the ECLSS.

It is thus imperative to provide an interactive environment for the crew model to study the mass flux values of the critical components under different scenarios. This was arranged by allowing the input variables like crew number, age, gender, height, weight and activity levels to be read from a user supplied data file.

As mentioned in chapter 2, the ECLSS consists of seven subsystems. The ECLSS modules are not water tight compartments but are closely connected to each other. The gaseous constituents and the water balances interact as:

- CO₂ reduction generates water which is processed to potable water
standards
• The bleed stream from the CO₂ removal process is directed to the trace removal subsystem.
• Humidity condensate from the humidity condensate removal subsystem is treated to obtain potable water.
• Part of the treated hygiene water is further treated to obtain makeup potable water and the remaining water is sent to the oxygen generation subsystem.
• The H₂ and O₂ from the O₂ generation subsystem are stored in their respective buffers
• The H₂ for CO₂ reduction is obtained from the H₂ buffer while the O₂ required for makeup O₂ in the cabin and for solid waste treatment is drawn from the O₂ buffer.

Figure illustrating the interaction between the LSS modules and the crew is included in Appendix 2.

In order to develop a complete ECLSS model which takes into account the interaction between the respective ECLSS subsystems, the ASPEN models of the different ECLSS subsystems (which were developed in chapter 4) were unified in a program. The user written crew model was then appended to the main program. The code is included in Appendix 3.

Mass Balance Sensitivity of the ECLSS

The utility of the type of LSS configuration chosen for the mission (e.g. open loop, partially closed loop, or completely closed loop) can be gauged by estimating the flow rates of the critical components. Figures 25A, 25B and 25C show the variation in mass flow rates for three cases where,
Case 1: Completely open loop.

Case 2: Partially closed loop. No O₂ generation and CO₂ reduction subsystems are present. Food is supplied.

Case 3: Completely closed loop, however food is supplied.

It is to be noted that all the values shown in Figures 25A, 25B and 25C are for the baseline case of one man operating at basal metabolic rate.

Figure 25A: O₂, N₂ and CO₂ requirement for the three cases.
Figure 25B:  Comparison of water requirement

Figure 25C:  Comparison of total untreated solid waste
It is observed from Figure 25A that with regard O₂ and CO₂ flow rates, the closed loop is advantageous to the other cases, as excess O₂ is produced which is stored in the buffer, and all the CO₂ which is produced is reduced within the system. The utility of the water regeneration subsystem is well illustrated in Figure 25B. The negative sign for the partially closed loop indicates that since there is no O₂ generation subsystem excess water is produced which is stored in the buffer. From Figure 25C it is evident that for a completely open loop the increase in amount of solid waste to be handled adds a greater burden to the system.

According to different crew missions the crew specification changes. Thereby, in order to arrive at a preliminary estimate of the sizing of the different ECLSS functional units and storage tank capacity, it is essential to know the mass-flow rates of the substreams within the ECLSS. Figures 26A, 26B and 26C show the variation in mass flow rates of the critical components with changing scenarios.

Case 1: 1 man operating at basal metabolic rate

Case 2: 1 man operating at 4 times the basal metabolic rate

Case 3: 1 man operating at 4 times basal metabolic rate and 1 woman operating at 3 times the basal metabolic rate.
Figure 26A: Comparison of food requirement for the three cases.

Figure 26B: Overall effect on the oxygen buffer for different crew scenarios.
Comparing Figures 26A, 26B and 26C we see that as the food consumption increases, the amount of O₂ and H₂ in the buffer increases. This is in accordance with the basic mass conservation law for a steady state system i.e. the excess oxygen and hydrogen atoms which enter the system in the form of food eventually leave the system as gases which are then stored in the buffer.

**Technology trade study**

As outlined in chapter 4 each ECLSS subsystem can be modeled by a wide variety of candidate technologies. In order to choose the appropriate technology for the ECLSS configuration desired apart from knowing the subsystem efficiency, it is also essential to know how the introduction of the system affects the rest of the LSS.
Figures 27 and 28 show the comparison between competing technologies for water regeneration and CO₂ reduction respectively.

Figure 27A: Mass flow rates of brine from the VCD and TIMES.

Figure 27B: Effect of VCD and TIMES on the overall system.
From Figures 27A, 27B, and 27C, we can deduce that VCD is more advantageous than TIMES, as:

- The cooling load required for VCD is lower than that required for TIMES. This is in accordance with the values reported in literature, which says that the TIMES requires twice the energy per pound of water produced compared to the VCD due to the inherent inefficiencies of the process, (Schubert, 1983).
- More O₂ and H₂ is stored in the buffer when a VCD is used.
- Recovery efficiency of the VCD is greater than that of the TIMES. This is in agreement with the reported values of 96% recovery for the VCD and 92% for the TIMES, (Schubert, 1983). The simulation values reported are however higher than the experimentally calculated values. This can be explained because for our simulation purpose we have assumed a steady state continuous process.
Figure 28A: Comparison of H$_2$ feed flow rate required for Bosch and Sabatier.

Figure 28B: Comparison of the overall effect on the entire system due to the presence of the Bosch or the Sabatier.
Similar to our deductions for VCD and TIMES from Figures 28A, 28B and 28C we can draw the analogy that the Bosch process is more advantageous as compared to the Sabatier process. This is substantiated by:

- The amount of H₂ required for the Sabatier being greater than the Bosch a continuous supply of H₂ is required to be supplied to the system for the operation of the Sabatier reactor. This is in agreement with the stoichiometric relation of the reactions taking place in the two reactors respectively.

- The amount of O₂ and H₂ stored in the buffer is greater when a Bosch reactor is used in place of a Sabatier reactor.

- The overall cooling load required is less for a Bosch as compared to a Sabatier. This is in apparent contradiction to the reported values in literature where the power requirement for a Bosch is 1.1 times that for the Sabatier, (Spina and Lee, 1985). This can be explained by considering that the amount of CO₂ to be treated for the basal case for the Bosch and Sabatier is 0.0465 and 0.4327 lb/hr respectively. This is so
because all the CH₄ formed in the Sabatier is oxidized in the Trace removal subsystem which adds an additional load to the CO₂ reduction subsystem.

Parameter sensitivity

The developed integrated model of the ECLSS can be used to study the performance criterion of the ECLSS functional units. Figure 29 shows the variation in performance of the Bosch reactor and its overall effect on the entire ECLSS as the mass ratio of the bleed stream from the Bosch to the Trace Removal system is varied from 0 to 1.0 as shown in the three cases, where:

Case 1: Bleed fraction is 0.0
Case 2: Bleed fraction is 0.5
Case 3: Bleed fraction is 1.0

Figure 29A: Comparison of efficiency for different bleed fractions.
Figure 29B: Comparison of cooling loads for different bleed fractions

Figure 29C: Overall effect on the system for the different bleed fractions adopted
The result shown in Figure 29A is in accordance with the values reported in literature (Spina and Lee, 1985). The single pass efficiency for a Bosch reactor using iron catalysts is reported to be less than 10%. The value calculated from our model (bleed fraction equal to 1.0) is 37.6%. This discrepancy could be explained by the over estimation in our model where Gibbs equilibrium is assumed to be attained in both the reactors. This does not occur under actual operating conditions, as mentioned earlier in chapter 4. With the total recycle, (bleed fraction equal to 0.0) conversion of 99.6% is attained which is again in accordance with Lee's results, (Spina and Lee, 1985).

The integrated ECLSS model can also be used to study the effect of varying the cabin conditions on the overall system. The result of our analysis can be used to determine an optimum between cabin conditions and amount of expendables to be supplied or stored. Two sample cases are observed for different environment specifications in the cabin. Both the cases are for one man operating at basal metabolic rate. The situations are as:

<table>
<thead>
<tr>
<th>Table 4: Different scenarios for cabin environment</th>
</tr>
</thead>
<tbody>
<tr>
<td>Mole fraction of gases in the cabin</td>
</tr>
<tr>
<td>Case 1</td>
</tr>
<tr>
<td>Case 2</td>
</tr>
<tr>
<td>Oxygen</td>
</tr>
<tr>
<td>Nitrogen</td>
</tr>
<tr>
<td>Water vapor</td>
</tr>
<tr>
<td>Carbon dioxide</td>
</tr>
</tbody>
</table>
Figure 30: Comparison of the effect on the overall LSS for different cabin environments.

As shown in Figure 30 Case 1 is optimum compared to Case 2 as more O2 and H2 is stored in the buffer. This shows that the optimum cabin requirement is Case 1 which is same as the standard requirements for the cabin.(Humphries et al, 1986)
The goal of any energy conservation study is to establish the minimum amount of energy necessary to operate a certain process. In the study which was carried out, (Brandhorst, 1991) the per capita power consumption for a partially closed and completely closed regenerable life support system were estimated as 3.5 kw/individual and 10 - 12 kw/individual respectively.

The objective of this chapter is to present an estimate of an alternative energy efficiency of the various sub units comprising the closed loop life support system calculated on the basis of the second law of thermodynamics.

**Why second law?**

In designing different LSS modules scientists usually consider the efficiency of the equipment based on the first law of thermodynamics. It has been shown in general that the average first law efficiency of most process is about 50%, (Cook, 1971).

However in a long term application we may be faced with the problem of entropy crisis. A system is using energy sources of low entropy probably solar and releasing energy of higher entropy. This is of concern to us because our goal should be to keep the net entropy production of a system as low as possible, and still ensure a positive entropy gradient between the system and the surroundings. As the net entropy production of a system increases, the entropy gradient between the system and the surrounding decreases and the system will gradually approach equilibrium with the surroundings until reaching the point where the entropy gradient is equal to zero. At this point no work can be extracted from the system. A measure of this
entropy generation can be obtained by evaluating the second law efficiency of the process.

From another perspective, from literature, the second law efficiency of a process is usually about 10%, (Reistad,1980). This indicates that 90% of the available work is usually lost in irreversibilities. Thus, the second law efficiency indicates that there is a great margin for improvement in the design of equipments auguring well for further large and rapid increases in efficiency.

Methodologies available for the second law analysis

The lowest energy we use can be derived from the application of the second law of thermodynamics. Quantities such as the available energy may be used to describe the irreversibilities of the process, (Denbigh, 1957). There are two widely used methods to study the process irreversibilities. They are:

- Availability analysis/Exergy analysis.
- Lost work analysis.

We shall now dwell briefly upon the steps followed for each of the above methodologies.

Availability Analysis

The technique of Availability Analysis was first proposed by Guoy and Stodola (Guoy, 1889 and Stodola 1898). Over the past years Availability Analysis has been extensively used in areas ranging from space heating to cryogenic processes, (Herbert and Stephen 1980, Ahern, 1980 (a)).

Many processes occur in an atmosphere which for all practical purposes is an inexhaustible source (or sink). The process is usually occurring because the system is not in thermal, mechanical or chemical equilibrium with the surroundings. The tendency of any natural process is to come to equilibrium with the surroundings.
This equilibrium state is known as the "dead state", (Moran, 1982). In the dead state there is no spontaneous change occurring within the system or the environment, or no spontaneous interaction between the two.

As the system moves towards its dead state it expends energy. Maximum work can be extracted if the system under study undergoes a totally reversible process in coming to its dead state. This maximum work is called as "available work". For an open system where there is inflow and outflow of mass, the available energy associated with mass inflow is termed as "exergy", (Moran, 1982).

The following general equation has been proposed to calculate the exergy at a point in the system (Ahern, 1980)

\[
\text{Exergy} = (U - U_0) - T_0 (S - S_0) + \frac{P_0}{J} (v - v_0) + \frac{v^2}{2gJ} + (z - z_0) \frac{g}{gcJ} + \Sigma c (\mu_c - \mu_0) W_c + E_i A_i R_i (3T^4 - T_0^4 - 4T_0 T_3) + \ldots
\]  

where:

- \((U - U_0)\) is the internal energy
- \(T_0 (S - S_0)\) is the entropy
- \(\frac{P_0}{J} (v - v_0)\) is the work
- \(\frac{v^2}{2gJ}\) is the momentum
- \((z - z_0) \frac{g}{gcJ}\) is work done against gravity
- \(\Sigma c (\mu_c - \mu_0) W_c\) is the chemical work
- \(E_i A_i R_i (3T^4 - T_0^4 - 4T_0 T_3)\) is radiation emission

the subscript 0 indicates the reference state or the environment

The thermodynamic efficiency based on exergetic analysis may be defined as, (Chiu and Newton, 1980)
Thermodynamic efficiency = \frac{\text{exergy in product}}{\text{exergy supplied to the system}} \quad (33)

**Lost Work Analysis**

The method of lost work analysis was first proposed by Seader, (Nevers and Seader, 1980).

The basic requirement of the second law is:

The total entropy change of an isolated system,

\[ \Delta S_{\text{sys}} \geq 0.0 \quad (34) \]

For a control volume with a steady state process where the surroundings are at a temperature of \( T_0 \), the total entropy change of the system is given by, (Smith and Van Ness 1987)

\[ \text{The rate of change of } = \text{ Net rate of entropy + Rate of entropy exchange with the surroundings} \]

Thus

\[ \Delta(Sm)_{fs} - \frac{Q}{T_0} > 0.0 \quad (35) \]

where \( \Delta(Sm)_{fs} \) is the difference in entropy between the feed and the product

\( Q \) is the net heat transfer from the system to the surroundings

For a steady state flow process there is no internal energy accumulation within the system. The law of conservation of energy can be expressed as,
\[ \Delta \left[ (H + \frac{1}{2} u^2 + zg) \right]_{fs} = Q - W_s \quad (36) \]

where \( \Delta \left[ (H + \frac{1}{2} u^2 + zg) \right]_{fs} \) is the difference in energy between the in flow and the outflow streams.

\( Q \) is the heat flow into the system

\( W_s \) is the work done by the system

For any system which requires work the amount of work required will be a minimum if the system undergoes a reversible change. This minimum work required is called the "ideal work", \( (W_{\text{ideal}}) \). Since there is no degradation of work the entropy generation for a reversible process is equal to zero and equation (35) becomes

\[ Q = T_0 \Delta(Sm)_{fs} \]

Substituting the above value in equation (36) and rearranging gives

\[ W_{\text{ideal}} = T_0 \Delta(Sm)_{fs} - \Delta \left[ (H + \frac{1}{2} u^2 + zg) \right]_{fs} \quad (37) \]

In most processes the kinetic and potential energy terms are negligible, and equation (37) can be written as

\[ W_{\text{ideal}} = T_0 \Delta(Sm)_{fs} - (\Delta H)_{fs} \quad (38) \]

It is justified to mention here that from an availability viewpoint, according to the definition of \( W_{\text{ideal}} \) as given in (38), the minimum work required is equivalent to the difference in availability between the input and the output streams.

For a work producing process the "lost work" is the work which is lost due to irreversibilities within the process. It is expressed as the difference between the ideal work which could be produced by the process \( (W_{\text{ideal}}) \) and the actual work
produced by the process ($W_s$). Thus from equations (36) and (38), $W_{\text{lost}}$ can be written as

$$W_{\text{lost}} = T_0 \Delta(S_{m})_{fs} - Q \quad (39)$$

Conventionally, there are two kinds of processes. A spontaneous process is one which produces work, i.e. $W_{\text{ideal}}$ is positive. Then

$$W_s = W_{\text{ideal}} - W_{\text{lost}} \quad (40)$$

A nonspontaneous process is one which requires some form of external work to be supplied, i.e. $W_{\text{ideal}}$ is negative. Hence

$$|W_s| = |W_{\text{ideal}}| + W_{\text{lost}} \quad (41)$$

Thereby the second law efficiency for each type of process can be defined as

$$\eta_2 \ (\text{spontaneous process}) = \frac{W_s}{W_{\text{ideal}}} \quad (42)$$

$$\eta_2 \ (\text{nonspontaneous process}) = \frac{W_{\text{ideal}}}{W_s} \quad (43)$$

For our analysis we used the Lost Work analysis method. The reasons for doing so are enumerated below:

- Calculation of lost work gives us a more intuitive insight into the energy loss within the process.
- For a process where the goal is to produce a stream having high availability, the calculation of a availability loss in the output stream makes no sense. However in the lost work approach we are only concerned with the irreversibilities within the process.

Method of Approach

The technique adopted to calculate the thermodynamic efficiency of the ECLSS is:

1. The process boundary is defined.
(2) Consistent heat and material balance data is obtained. This data is obtained from our ASPEN model of the ECLSS.

(3) Selection of ambient temperature as a reference. We choose the cabin temperature of 70° F.

(4) Evaluating the $W_{\text{lost}}$ within each functional unit of the ECLSS subsystem.

(5) Calculating the $W_{\text{ideal}}$ for each subsystem based on the subsystem information.

(6) Knowing the $W_{\text{lost}}$ and $W_{\text{ideal}}$ the second law efficiency of each subsystem is evaluated.

Results

The advantage of the lost work analysis was demonstrated by applying this technique to a few ECLSS sample technologies. The stream information like enthalpy, entropy, and mass flow rates were obtained from our integrated ASPEN model. The values are obtained for a basal case of a crew of 1 man of 25 years of age weighing 60 kg operating at basal metabolic rate.

Wet Oxidation of Solid Waste

Refer Figure 24 for stream identification.
Table 5: Properties of streams involved in wet oxidation process.

<table>
<thead>
<tr>
<th>St</th>
<th>Stream</th>
<th>Mass flow rate (lb/hr)</th>
<th>Total enthalpy (Btu/hr)</th>
<th>Total entropy (Btu/hr °R)</th>
</tr>
</thead>
<tbody>
<tr>
<td>S1</td>
<td>3.44x10^-4</td>
<td>12.91x10^-4</td>
<td>23.56x10^-7</td>
<td></td>
</tr>
<tr>
<td>S2</td>
<td>9.16x10^-4</td>
<td>-1.4x10^-3</td>
<td>-2.64x10^-3</td>
<td></td>
</tr>
<tr>
<td>S3</td>
<td>1.26x10^-3</td>
<td>-4.77</td>
<td>-3.63x10^-6</td>
<td></td>
</tr>
<tr>
<td>S4</td>
<td>1.26x10^-3</td>
<td>-4.85</td>
<td>1.72x10^-5</td>
<td></td>
</tr>
<tr>
<td>S5</td>
<td>0.0</td>
<td>0.0</td>
<td></td>
<td></td>
</tr>
<tr>
<td>Q1</td>
<td>Heat Duty=</td>
<td>-4.77 Btu/hr</td>
<td></td>
<td></td>
</tr>
<tr>
<td>Q2</td>
<td>Heat Duty=</td>
<td>-0.08 Btu/hr</td>
<td></td>
<td></td>
</tr>
</tbody>
</table>

The lost work analysis of each unit operation block comprising the solid waste oxidizer is given in Table 6.

Table 6: Lost work estimation of the solid waste oxidizer

<table>
<thead>
<tr>
<th>Unit Operation block</th>
<th>Lost work (Btu/hr)</th>
<th>% of total lost work</th>
</tr>
</thead>
<tbody>
<tr>
<td>Wet oxidizer</td>
<td>4.71</td>
<td>98.1</td>
</tr>
<tr>
<td>Depressurizer</td>
<td>0.091</td>
<td>1.9</td>
</tr>
<tr>
<td>Total lost work</td>
<td>4.801</td>
<td></td>
</tr>
</tbody>
</table>

Wideal = 530 (1.72 x 10^-5 + 2.64 x 10^-3 - 23.56 x 10^-7) - (-4.85 -
12.91 \times 10^{-4} + 14.05 \times 10^{-4})
= 6.26 \text{ Btu/hr}

Wideal being positive this is a spontaneous process. Hence

\[ W_s = 6.26 - 4.801 = 1.459 \]

\[ \eta_2 = \frac{1.459}{6.26} \]

= 23.3%

**Catalytic Oxidation of Trace Contaminant Gases:**

Refer Figure 15 for stream identification.

### Table 7: Stream properties of Catalytic oxidation subsystem

<table>
<thead>
<tr>
<th>Stream</th>
<th>Mass flow rate (lb/hr)</th>
<th>Total enthalpy (Btu/hr)</th>
<th>Total entropy (Btu/hr ° R)</th>
</tr>
</thead>
<tbody>
<tr>
<td>S1</td>
<td>35.66</td>
<td>-3291.38</td>
<td>1.17</td>
</tr>
<tr>
<td>S2</td>
<td>3.45</td>
<td>-318.44</td>
<td>0.113</td>
</tr>
<tr>
<td>S3</td>
<td>3.45</td>
<td>135.9</td>
<td>0.71</td>
</tr>
<tr>
<td>S4</td>
<td>3.45</td>
<td>309.64</td>
<td>0.86</td>
</tr>
<tr>
<td>S5</td>
<td>3.45</td>
<td>309.33</td>
<td>0.86</td>
</tr>
<tr>
<td>S6</td>
<td>3.45</td>
<td>-145.0</td>
<td>0.39</td>
</tr>
<tr>
<td>S7</td>
<td>32.21</td>
<td>-2972.98</td>
<td>1.05</td>
</tr>
<tr>
<td>S8</td>
<td>32.21</td>
<td>-2975.56</td>
<td>1.05</td>
</tr>
<tr>
<td>S9</td>
<td>35.66</td>
<td>-3120.25</td>
<td>1.49</td>
</tr>
<tr>
<td>Q1</td>
<td>Heat duty = 173.7</td>
<td>(Btu/hr)</td>
<td></td>
</tr>
<tr>
<td>Q2</td>
<td>Heat duty = -0.297</td>
<td>(Btu/hr)</td>
<td></td>
</tr>
<tr>
<td>Q3</td>
<td>Heat duty = -2.56</td>
<td>(Btu/hr)</td>
<td></td>
</tr>
</tbody>
</table>
The lost work analysis of each unit operation block comprising the Catalytic Oxidizer is shown in Table 8.

Table 8: Lost work estimation of Catalytic oxidation subsystem

<table>
<thead>
<tr>
<th>Unit operation block</th>
<th>Lost work (Btu/hr)</th>
<th>% of total lost work</th>
</tr>
</thead>
<tbody>
<tr>
<td>Diverter</td>
<td>0.0</td>
<td>0.0</td>
</tr>
<tr>
<td>Heat exchanger</td>
<td>67.31</td>
<td>38.2</td>
</tr>
<tr>
<td>Heater</td>
<td>79.5</td>
<td>45.1</td>
</tr>
<tr>
<td>HTCO</td>
<td>0.297</td>
<td>0.17</td>
</tr>
<tr>
<td>LTCO</td>
<td>2.56</td>
<td>1.45</td>
</tr>
<tr>
<td>Mixer</td>
<td>26.5</td>
<td>15.04</td>
</tr>
<tr>
<td>Total lost work</td>
<td>176.17</td>
<td></td>
</tr>
</tbody>
</table>

Wideal = 530 (1.49 - 1.17) - (-3120.25 + 3291.38)

= -1.53 Btu/hr

This is a non spontaneous process. Hence

Ws = 1.53 + 176.17

= 177.7 Btu/hr

η2 = \frac{1.53}{177.7}

= 0.86%

**Bosch subsystem**

Refer Figure 19 for stream identification. The bleed stream fraction is assumed to be zero.
Table 9: Stream properties of Bosch subsystem

<table>
<thead>
<tr>
<th>Stream</th>
<th>Mass flw rate (lb/hr)</th>
<th>Total enthalpy (Btu/hr)</th>
<th>Total entropy (Btu/hr °R)</th>
</tr>
</thead>
<tbody>
<tr>
<td>S1</td>
<td>0.048</td>
<td>-183.79</td>
<td>-6.04×10^-4</td>
</tr>
<tr>
<td>S2</td>
<td>4.37×10^-3</td>
<td>-54.16 ×10^-6</td>
<td>-5.98×10^-7</td>
</tr>
<tr>
<td>S3</td>
<td>0</td>
<td>-677.1</td>
<td>-0.139</td>
</tr>
<tr>
<td>S4</td>
<td>0.398</td>
<td>515.04</td>
<td>0.778</td>
</tr>
<tr>
<td>S5(solid)</td>
<td>1.3× 10^-2</td>
<td>4.51</td>
<td>4.27×10^-3</td>
</tr>
<tr>
<td>S5(vapor)</td>
<td>0.385</td>
<td>-130.4</td>
<td>0.436</td>
</tr>
<tr>
<td>S6</td>
<td>3.9× 10^-2</td>
<td>-268.11</td>
<td>-0.087</td>
</tr>
<tr>
<td>S7(solid)</td>
<td>1.3× 10^-2</td>
<td>-0.08</td>
<td>-1.5 ×10^-4</td>
</tr>
<tr>
<td>S7(vapor)</td>
<td>0.346</td>
<td>-507.11</td>
<td>-0.177</td>
</tr>
<tr>
<td>S8</td>
<td>1.3× 10^-2</td>
<td>-0.015</td>
<td>-2.87×10^-5</td>
</tr>
<tr>
<td>S9</td>
<td>0.398</td>
<td>-677.1</td>
<td>-0.139</td>
</tr>
<tr>
<td>Q1</td>
<td>Heat duty =</td>
<td>1192.14</td>
<td>Btu/hr</td>
</tr>
<tr>
<td>Q2</td>
<td>Heat duty =</td>
<td>-640.93</td>
<td>Btu/hr</td>
</tr>
<tr>
<td>Q3</td>
<td>Heat duty =</td>
<td>-649.41</td>
<td>Btu/hr</td>
</tr>
</tbody>
</table>

Table 10 indicates the amount of lost work in each functional unit comprising the Bosch subsystem.
Table 10: Lost work analysis of unit operation blocks of Bosch subsystem

<table>
<thead>
<tr>
<th>Unit operation block</th>
<th>Lost work (Btu/hr)</th>
<th>% lost work</th>
</tr>
</thead>
<tbody>
<tr>
<td>Mixer</td>
<td>6.89</td>
<td>0.55</td>
</tr>
<tr>
<td>Reactor 1</td>
<td>486.01</td>
<td>39.04</td>
</tr>
<tr>
<td>Reactor 2</td>
<td>461.95</td>
<td>37.11</td>
</tr>
<tr>
<td>Condensate remover</td>
<td>276.13</td>
<td>22.18</td>
</tr>
<tr>
<td>Solid carbon remover</td>
<td>13.78</td>
<td>1.11</td>
</tr>
<tr>
<td>Total lost work</td>
<td>1244.76</td>
<td></td>
</tr>
</tbody>
</table>

\[
W_{\text{ideal}} = 530 \left(-2.87 \times 10^{-5} - 0.087 + 6.21 \times 10^{-4} + 5.98 \times 10^{-7} - 1.72 \times 10^{-5}\right) - \\
\left(-286.11 - 0.015 + 178.94 + 54.16 \times 10^{-6} + 4.85\right) \\
= 38.53 \text{ Btu/hr}
\]

Hence this is a spontaneous process.

\[
W_s = 38.53 - 1244.76 = -1206.23
\]

This is an example of a highly non ideal process. The actual process is spontaneous however under simulated working conditions the process requires external work to be supplied.

**Static Feed Water Electrolysis**

Refer Figure 23 for stream identification.
Table 11: Properties of streams present in the Water Electrolysis Subsystem.

<table>
<thead>
<tr>
<th>Stream</th>
<th>Mass flow rate (lb/hr)</th>
<th>Total enthalpy (Btu/hr)</th>
<th>Total entropy (Btu/hr °R)</th>
</tr>
</thead>
<tbody>
<tr>
<td>S1</td>
<td>$9.6 \times 10^{-2}$</td>
<td>-654.8</td>
<td>-0.21</td>
</tr>
<tr>
<td>S2</td>
<td>$9.6 \times 10^{-2}$</td>
<td>-654.83</td>
<td>-0.21</td>
</tr>
<tr>
<td>S3</td>
<td>$9.6 \times 10^{-2}$</td>
<td>-654.83</td>
<td>-0.21</td>
</tr>
<tr>
<td>S4</td>
<td>$9.6 \times 10^{-2}$</td>
<td>-0.38</td>
<td>-0.27</td>
</tr>
<tr>
<td>S5</td>
<td>$1.07 \times 10^{-2}$</td>
<td>-0.25</td>
<td>-4.76 \times 10^{-4}</td>
</tr>
<tr>
<td>S6</td>
<td>$8.52 \times 10^{-2}$</td>
<td>-0.13</td>
<td>-2.46 \times 10^{-4}</td>
</tr>
<tr>
<td>Q1</td>
<td>Heat duty = -0.03 Btu/hr</td>
<td></td>
<td></td>
</tr>
<tr>
<td>Q2</td>
<td>Heat duty = 654.55 Btu/hr</td>
<td></td>
<td></td>
</tr>
</tbody>
</table>

Table 12: Lost work analysis within the Water Electrolysis Subsystem

<table>
<thead>
<tr>
<th>Unit operation block</th>
<th>Lost work (Btu/hr)</th>
<th>% lost work</th>
</tr>
</thead>
<tbody>
<tr>
<td>Pump</td>
<td>0.427</td>
<td>0.38</td>
</tr>
<tr>
<td>Cooler</td>
<td>0.03</td>
<td>0.03</td>
</tr>
<tr>
<td>Electrolyzer</td>
<td>96.99</td>
<td>87.08</td>
</tr>
<tr>
<td>Gas separator</td>
<td>13.93</td>
<td>12.51</td>
</tr>
<tr>
<td>Total lost work</td>
<td>111.38</td>
<td></td>
</tr>
</tbody>
</table>
Wideal = 530 (-4.76×10^-4 - 2.46×10^-4 + 0.21) - (-0.13 - 0.25 + 654.8)
= -543.5 Btu/hr

Wideal being negative, this is a spontaneous process.

\[ W_s = -654.88 \text{ Btu/hr} \]
\[ \eta_2 = \frac{543.5}{654.88} \]
= 82.99%

**Humidity Condensate Removal**

Refer Figure 16 for stream identification.

Table 13: Stream properties of streams occurring in Humidity Condensate Remover

<table>
<thead>
<tr>
<th>Stream</th>
<th>Mass flow rate (lb/hr)</th>
<th>Total enthalpy (Btu/hr)</th>
<th>Total entropy (Btu/hr ° R)</th>
</tr>
</thead>
<tbody>
<tr>
<td>S1</td>
<td>35.66</td>
<td>-3120.22</td>
<td>1.49</td>
</tr>
<tr>
<td>S2</td>
<td>7.18×10^-2</td>
<td>-489.58</td>
<td>-0.154</td>
</tr>
<tr>
<td>S3</td>
<td>35.59</td>
<td>-2783.35</td>
<td>1.35</td>
</tr>
<tr>
<td>Q1</td>
<td>Heat duty =</td>
<td>-152.71</td>
<td>Btu/hr</td>
</tr>
</tbody>
</table>

Since the heat of condensation is not being used to do any useful work, hence,

Lost work = 530 (1.35 + 0.154 - 1.49) + 152.71
= 160.13 Btu/hr

Wideal = 530(1.35 + 0.154 - 1.49) - (-489.58 - 2783.35 + 3120.22)
= 160.13 Btu/hr
Thus $r_2 = 0$

This is valid since we had assumed that the heat of condensation is not used for doing useful work, thereby the amount of work which can be obtained from the process is zero.
We arrive at the following conclusions from the first part of this project:

The integrated Environmental Control and Life Support System (EELS) model can be used to assess the parameter requirements for different space station conditions. The utility of a particular configuration is judged on the basis of the amount of oxygen and hydrogen stored in their respective buffers and the heating/cooling load required. The excess oxygen and hydrogen is stored as a reserve for cases of exigency for example to provide oxygen to the cabin if there occurs a failure in the oxygen generation unit. However the oxygen and hydrogen stored in the buffer will bear an impact on the load carrying capability of the space shuttle. This has not been explored in our present model.

Recommended future work in this area is:

- To develop a more rigorous model of the various EELS components like the water electrolysis system and the CO2 membrane removal system.
- To develop a rigorous dynamic biological model of the crew.
- To develop a dynamic model of the entire ECLSS using the values obtained from our steady state model as the initial values.

The latter part of this project was concerned with the ‘lost work’ analysis. An analysis of lost work at the subsystem component level enables the designer to identify principal sources of entropy generation and strive to minimize them. Thus the overall conclusion to be drawn from this work is that there exists a potential for energy conservation in the currently used ECLSS technologies which warrants evaluation of the present operating conditions. In order to arrive at a detailed
optimum design of the ECLSS subsystem it is recommended to evaluate the lost work for every hardware component constituting the subsystem which would enable the designer to exactly pinpoint the areas of maximum entropy generation.


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APPENDIX 1
DETAILED DERIVATION OF EQUATIONS USED FOR THE COMPARTMENT MODEL OF THE HUMAN RESPIRATORY TRACT

Upper Respiratory Tract

Ideal gas law holds good. Therefore:
\[
\frac{Q_1}{70 + 460} = \frac{Q_2}{98.6 + 460}
\]  \hspace{1cm} (A1)

Lower Respiratory Tract

Volume of water entering the compartment = \(\frac{15}{760 - 15}\) = 0.021 lit/lit of dry gas

Volume of water leaving the compartment = \(\frac{47}{760 - 47}\) = 0.066 lit/lit of dry gas

Volume of dry gas entering = \(Q_2 - 0.021 \times Q_2\) = 0.98 \(\times Q_2\) lit

Total volume of water present in outgoing gas = \(0.066 \times 0.98 \times Q_2\) = 0.065 \(\times Q_2\) lit

Thereby total volume of gas leaving the compartment (Q3) is given by :
\[
Q_3 = (0.065 + 0.98) \times Q_2
\]  \hspace{1cm} (A2)

Since no exchange of N₂, CO₂ or O₂ occurs in the lower respiratory tract.

Therefore the outgoing mole fractions of O₂, CO₂ and N₂ will be given by
\[
Y_x^i \times Q_2 = Y_x^o \times Q_3
\]  \hspace{1cm} (A3)

where
\[
Y_x^i \text{ mole fraction of species } x \text{ (may be O₂, CO₂ or N₂) in } Q_2
\]
\[
Y_x^o \text{ mole fraction of species } x \text{ in } Q_3
\]
Lungs:

Solving equations (21) to (24) as given in chapter 3

\[ Q_4 = \left( Q_3 \times (Y_{O_2}^{i} + Y_{CO_2}^{i} + Y_{N_2}^{i}) \right) - VO_2 + VCO_2 / 0.938 \]  
(A4)

The concentration of the outgoing gas from the lungs is calculated using the following equations:

\[ Y_{O_2} = \frac{(Q_3 \times Y_{O_2}^{i}) - VO_2}{Q_4} \]  
(A5)

\[ Y_{CO_2} = \frac{(Q_3 \times Y_{CO_2}^{i}) + VCO_2}{Q_4} \]  
(A6)

\[ Y_{N_2} = \frac{(Q_3 \times Y_{N_2}^{i})}{Q_4} \]  
(A7)

Q1, Q2, Q3, Q4, VO2, VCO2, VE, YO2, YN2 and YCO2 are as defined in chapter 3.

MO2 the volume of O2 required for metabolism is related to VO2 by,

\[ MO_2 = \left( \frac{VO_2}{60.0} \right) \times \left( \frac{273}{310} \right) \]  
(A8)

and

\[ Q_3 = VE \times 60.0 \]  
(A9)

The total mass flow rates in lb/hr of component i (where i can be O2, N2, CO2 or H2O) in expired air (M_{i^{exp}}) and in inspired air (M_{i^{ins}}) is given by:

\[ M_{i^{exp}} = \left( Q_4 \times Y_i \times \text{molecular weight of } i \times 0.035 \right) / (0.7302 \times 558.6) \]  
(A10)

\[ M_{i^{ins}} = \left( Q_4 \times Y_i \times \text{molecular weight of } i \times 0.035 \right) / (0.7302 \times 530.0) \]  
(A11)
APPENDIX 2

BLOCK FLOW DIAGRAM OF THE INTEGRATED ECLSS MODEL WITH THE CREW

The integrated ASPEN model of the ECLSS is included here. Man is shown as a regular block.
NOMENCLATURE:

A  WATER PRESENT IN FOOD
B  WATER GENERATED DUE TO METABOLISM OF FOOD
C  HUMIDITY CONDENSATE
D  CONDENSATE WATER FROM $\mathbf{CO}_2$ REDUCTION
E  WATER USED FOR OXYGEN GENERATION
F  SWEAT GENERATED BY THE CREW
G  EXCESS WATER USED FOR SATURATING THE EXPIRED AIR
H  MAKEUP OXYGEN FROM THE BUFFER TO THE CABIN
I  EXPIRED AIR FROM THE CREW
J  AIR LEAK FROM THE CABIN
K  AIR INSPIRED BY THE CREW
L  TRACE CONTAMINANT STREAM
M  BLEED STREAM FROM THE CARBON DIOXIDE REDUCTION PROCESS
P  SOLID CARBON GENERATED IN THE BOSCH REACTOR
NOMENCLATURE CONTINUED:

- $Q$  UNTREATED SOLID WASTE
- $R$  SOLID WASTE FROM CREW TO BE TREATED
APPENDIX 3

THIS IS AN ASPEN SOLUTION CONSIDERING THE MAN IN THE CABIN
; APPENDING EACH ECLSS MODULE TO THE CABIN BLOCK

; SPECIFYING THE UNIT SYSTEM FOR INPUT INFORMATION
IN-UNITS ENG
; SPECIFYING THE UNIT SYSTEM FOR OUTPUT INFORMATION
OUT-UNITS ENG
; DEFINE COMPONENTS THAT ARE IN THE SYSTEM
COMPONENTS C1 WATER/CO2 C02 /C3 HYDROGEN/C4 NITROGEN/C5 OXYGEN/C6 CH4
COMPONENTS C7 CARBON-MONOXIDE/C9 ACETYLENE/C12 C
COMPONENTS C13 ETHYLENE/C14 ETHANE/C2 UREA
DATABANKS SOLIDS/DIPPRPCD
; SPECIFYING PHYSICAL PROPERTY OPTIONS
PROPERTIES SYSOP0

; FLOWSHEET SECTION OF ASPEN INPUT FILE
; THIS SECTION IS FOR WATER ASPEN REGENERATION SUBSYSTEM

FLOWSHEET WATERREGENERATION
   BLOCK HEAT11 IN=METW PW01 FOODW    OUT=HE11OUT
   BLOCK WASEP IN=HE11OUT    OUT=WAEX1
   BLOCK W1 IN=URINE    OUT=URINE1
   BLOCK MIX12 IN=URINE1 S11 FD1U OUT=FD1OUT
   BLOCK MIX18 IN=S2 MAKEUPS2 OUT=S21
   BLOCK MIX15 IN=S21 SS2 OUT=FD2OUT
   BLOCK MIX13 IN=FD1OUT FD2OUT OUT=F1
   BLOCK SPTT2 IN=CON1 OUT=CON2 CONR
   BLOCK STILL IN=FOUT OUT=CON BRINE
   BLOCK P1 IN=F1 CONR OUT=FOUT
   BLOCK P4 IN=CON OUT=CON1
   BLOCK P5 IN=CON2 OUT=CON3
   BLOCK SPTT3 IN=CON3 OUT=S1 S2 S3
   BLOCK SPTT4 IN=S3 OUT=S3OUT MAKEUPPOT
   BLOCK MIX20 IN=S1 URFLUSH OUT=S11
   BLOCK W2 IN=W1 OUT=WA1
   BLOCK POTMIX IN=MAKEUPPOT DECAN WA1 OUT=PWO
   BLOCK POTSEP IN=PWO OUT=PWO1 PWOEX

; THIS SECTION IS FOR TRACE REMOVAL, DEHUMIDIFICATION, CO2 REMOVAL, AND PART OF CO2 REDUCTION
; FLOWSHEET INITIAL
FLOWSHEET INITIAL
   BLOCK CAB1 IN=N202 CABCO2 CABH20 EXHAIR SW OUT=CAB1OUT
   BLOCK CAB2 IN=CAB1OUT OUT=AIRLEAK INSAIR AIROUT
   BLOCK MIX2 IN=AIROUT TC BLEED1 EX OUT=OUTM
   BLOCK SPLIT1 IN=OUTM OUT=LT HT
   BLOCK HEEX1 IN=HT HEIN OUT=HEOUT2 HEATOUT
   BLOCK HEA6 IN=HEATOUT OUT=HEA6OUT
   BLOCK GIBB1 IN=HEA6OUT OUT=HEIN
   BLOCK GIBB2 IN=LT OUT=GIBBOUT
   BLOCK MIXT IN=GIBBOUT HEOUT2 OUT=HUHXOUT
   BLOCK HEA2 IN=HUHXOUT OUT=CAIR WA
   BLOCK SEP3 IN=CAIR OUT=ABAIR1 SEPC EX
BLOCK COOL IN=ABAIR1 OUT=ABAIR
BLOCK MIX5 IN=SEPC SEPOUT2 HYDRO WASCO21 OUT=MIX5OUT
BLOCK REAC1 IN=MIX5OUT OUT=REA1

; THIS SECTION IS FOR REMAINING CO2 REDUCTION SUBSYSTEM
; INVOLVING SOLID CARBON AND FOR THE SOLID WASTE INCINERATOR
; SUBSYSTEM

FLOWSHEET MIDDLE
BLOCK CHNGR1 IN=REA1 OUT=REA1OUT
BLOCK REAC2 IN=REA1OUT OUT=REA2
BLOCK HE1 IN=REA2 OUT=HE1OUT DECAN
BLOCK SEPA IN=HE1OUT OUT=SEPOUT SOLIDC
BLOCK SEPA1 IN=SEPOUT OUT=SEPOUT1 BLEED
BLOCK SOLIDINC IN=SOLIDW WASO2 OUT=WASCO2A
BLOCK SOLP IN=WASCO2A OUT=WASCO2 WASC

; THIS SECTION INCLUDES THE O2 GENERATION SUBSYSTEM

FLOWSHEET FINAL
BLOCK CHNGR3 IN=SEPOUT1 OUT=SEPOUT2
BLOCK PSFE IN=REMWATER S31 PWOEX1 OUT=PSFE0
BLOCK HSFE IN=PSFE0 OUT=HSFE0
BLOCK STOIC1 IN=HSFE0 OUT=STOIC1OUT
BLOCK SEPSFE IN=STOIC1OUT OUT=HYDI OXYGI REMWATER
BLOCK CHNGR7 IN=S3OUT OUT=S31
BLOCK CHNGR8 IN=WASCO2 OUT=WASCO21
BLOCK CHNGR9 IN=BLEED OUT=BLEED1
BLOCK CHNGR10 IN=PWOEX OUT=PWOEX1

; SPECIFY FEED STREAMS

DEF STREAMS MIXCISLD MIDDLE WATERREGENERATION

STREAM EXHAIR PRES=1[ATM] TEMP=98.6 NPHASE=1
  MASS-FLOW C1 0.5/C02 0.5/C4 3.0/C5 2.0
STREAM TC PRES=1[ATM] TEMP=70.0 MASS-FLOW=3.27D-4
  MASS-FRAc C7 0.84/C6 0.084/C14 0.00056/C13 0.000014/C9 0.00014/C3 0.075
STREAM HYDRO PRES=1[ATM] TEMP=70.0
  MASS-FLOW C3 100.0
STREAM FD1U
  SUBSTREAM CISOLID TEMP=98.6 PRES=1[ATM]
    MASS-FLOW C2 0.001
STREAM SS2
  SUBSTREAM CISOLID TEMP=70 PRES=1[ATM]
    MASS-FLOW C2 1D-5
STREAM METW TEMP=98.6 PRES=1[ATM]
  MASS-FLOW C1 0.01
STREAM MAKEUPS2 PRES=1[ATM] TEMP=70.0
  MASS-FLOW C1 10.0
STREAM URFPLUSH TEMP=70.0 PRES=1[ATM]
  MASS-FLOW C1 100.0
STREAM CABCO2 TEMP=69.0 PRES=1[ATM] NPHASE=1 PHASE=V
  MASS-FLOW C02 100.0
STREAM CABH2O TEMP=69.0 PRES=1[ATM] NPHASE=1 PHASE=V
  MASS-FLOW C1 100.0
STREAM N202 TEMP=69.0 PRES=1[ATM] NPHASE=1 PHASE=V
MASS-FLOW C4 100.0/C5 100.0
STREAM SOLIDW
    SUBSTREAM CISOLID PRES=1[ATM] TEMP=98.6
    MASS-FLOW C12 100.0
STREAM WAS02 TEMP=70 PRES=1[ATM]
    MASS-FLOW C5 100.0
STREAM FOODW TEMP=70 PRES=1[ATM]
    MASS-FLOW C1 100.0
STREAM SW TEMP=98.6 PRES=1[ATM] NPHASE=1 PHASE=V
    MASS-FLOW C1 100.0

; DESCRIPTION OF ALL THE BLOCKS INCLUDED IN THE PROGRAM

; BLOCKS FOR VCD AND MAN WATER BALANCE
BLOCK HEAT11 HEATER
    PARAM TEMP=98.6 PRES=1[ATM]
BLOCK WASEP FSPLIT
    PARAM PRES=1[ATM]
    MASS-FLOW WAEXH 0.5/SW1 0.6
BLOCK MIX12 MIXER
BLOCK MIX15 MIXER
BLOCK MIX13 MIXER
    PARAM PRES=20.0
BLOCK SPTT2 FSPLIT
    FRAC CONR 0.6
BLOCK STILL SEP
    FRAC STREAM=BRINE SUBSTREAM=MIXED COMPS=C1 FRACS=0.8
    FRAC STREAM=BRINE SUBSTREAM=CISOLID COMPS=C2 FRACS=1.0
    FLASH-SPECS CON TEMP=70 PRES=0.5 NPHASE=1 PHASE=L
    FLASH-SPECS BRINE TEMP=70 PRES=0.5 NPHASE=1 PHASE=L
BLOCK P1 PUMP
    PARAM PRES=0.5 NPHASE=2 MAXIT=60 TOL=0.01
BLOCK P4 PUMP
    PARAM PRES=20.0 NPHASE=2 MAXIT=60 TOL=0.01
BLOCK P5 PUMP
    PARAM PRES=14.7 NPHASE=2 MAXIT=60 TOL=0.01
BLOCK SPTT3 FSPLIT
    MASS-FLOW S1 0.01/S2 2.3
BLOCK SPTT4 FSPLIT
    PARAM PRES=1[ATM] NPHASE=1 PHASE=L
    FRAC MAKEUPROT 0.5
BLOCK POTMIX MIXER
BLOCK POTSEP FSPLIT
    PARAM PRES=1[ATM]
    MASS-FLOW PW01 0.1
BLOCK MIX18 MIXER
BLOCK MIX20 MIXER
BLOCK CHNGR7 CLCHNG
BLOCK CHNGR10 CLCHNG

; BLOCKS FOR CABIN
BLOCK CAB1 MIXER
BLOCK CAB2 FSPLIT
    PARAM PRES=1[ATM] NPHASE=1
MASS-FLOW AIRLEAK 6.5D-3
R-FRAC AIROUT 0.98

; BLOCKS FOR TRACE REMOVAL SUBSYSTEM
BLOCK MIX2 MIXER
   PARAM PRES=1[ATM]
BLOCK SPLIT1 FSPLIT
   PARAM PRES=1[ATM]
   FRAC LT 0.7
BLOCK HEEXI HEATX
   PARAM T-COLD=600
   FEEDS HOT=HEIN COLD=HT
   PRODUCTS HOT=HEOUT2 COLD=HEATOUT
BLOCK HEA6 HEATER
   PARAM TEMP=795.0
BLOCK GIBB1 RGIBBS
   PARAM TEMP=795 PRES=1[ATM]
   PROD C1/C02/C4/C5/C3/C7/C6
BLOCK GIBB2 RGIBBS
   PARAM TEMP=70 PRES=1[ATM] MAXIT=100 TOL=1D-2
   PROD CO2/C1/C4/C5/C7/C6/C3
BLOCK MIXT MIXER
   PARAM PRES=1[ATM]

; BLOCKS FOR DEHUMIDIFICATION
BLOCK HEA2 SEP
   PARAM PRES=1[ATM]
   FRAC STREAM=CAIR COMPS=C1 CO2 C4 C5 C6 C7 C3 FRACS=0.8 1.0 1.0 &
   1.0 1.0 1.0 1.0
   FLASH-SPECS WA TEMP=40 PRES=1[ATM] NPHASE=1 PHASE=L
   FLASH-SPECS CAIR TEMP=40 PRES=1[ATM] NPHASE=1 PHASE=V
BLOCK COOL HEATER
   PARAM TEMP=69.0 PRES=1[ATM]

; BLOCK FOR CO2 REMOVAL
BLOCK SEP3 SEP
   FRAC STREAM=ABAIR1 COMPS=CO2 C1 C4 C5 FRACS=0.5 1.0 1.0 1.0
   FRAC STREAM=EX COMPS=C6 C7 C3 CO2 FRACS=1.0 1.0 1.0 0.0
   FLASH-SPECS ABAIR1 TEMP=70.0 PRES=1[ATM]
   FLASH-SPECS SEPC TEMP=70.0 PRES=1[ATM]
   FLASH-SPECS EX TEMP=70.0 PRES=1[ATM]

; BLOCKS FOR CO2 REDUCTION USING BOSCH
BLOCK MIX5 MIXER
   PARAM PRES=1[ATM]
BLOCK REAC1 RGIBBS
   PARAM TEMP=1300[K] PRES=1[ATM]
   PROD CO2/C3/C7/C1/C6
BLOCK CHNGR1 CLCHNG
BLOCK REAC2 RGIBBS
   PARAM TEMP=900[K] PRES=1[ATM]
   PROD CO2/C3/C7/C6/C1/C12 S
BLOCK HE1 SEP
   FRAC STREAM=HE1OUT SUBSTREAM=MIXED COMPS=C1 CO2 C3 C7 C6 &
   FRACS=0.0 1.0 1.0 1.0 1.0
FRAC STREAM=HEIOUT SUBSTREAM=CISOLID COMPS=C12 FRACS=1.0
FLASH-SPECS DECAN TEMP=40 PRES=1[ATM] NPHASE=1 PHASE=L
FLASH-SPECS HEIOUT TEMP=40 PRES=1[ATM] NPHASE=1 PHASE=V

BLOCK SEPA SEP
FRAC STREAM=SEPOUT COMPS=CO2 C3 C7 C6 FRACS=1.0 1.0 1.0 1.0
FLASH-SPECS SOLIDC TEMP=70 PRES=1[ATM] NPHASE=1 PHASE=S
FLASH-SPECS SEPOUT TEMP=70 PRES=1[ATM] NPHASE=1 PHASE=V 1.0

BLOCK SEPA1 FSPLIT
PARAM PRES=1[ATM]
FRAC BLEED 0.0

BLOCK CHNGR3 CLCHNG

; BLOCKS FOR O2 GENERATION BY WATER ELECTROLYSIS
BLOCK PSFE PUMP
PARAM PRES=30.0

BLOCK HSFE HEATER
PARAM PRES=30.0 TEMP=70.0

BLOCK STOIC1 RSTOIC
PARAM TEMP=70.0 PRES=150.0
STOIC 1 MIXED C1 -1/C3 1/C5 0.5
CONV 1 MIXED C1 1.0

BLOCK SEPSFE SEP
PARAM PRES=14.7
FRAC STREAM=HYD1 COMPS=C3 FRACS=1.0
FRAC STREAM=OXYG1 COMPS=C5 FRACS=1.0
FLASH-SPECS HYD1 PRES=14.7 TEMP=70.0
FLASH-SPECS OXYG1 PRES=14.7 TEMP=70.0
FLASH-SPECS REMWATER PRES=14.7 TEMP=70.0

; BLOCKS FOR SOLID WASTE INCINERATOR
BLOCK SOLIDINC RSTOIC
PARAM TEMP=350 PRES=1067.0
STOIC 1 CISOLID C12 -1/MIXED C5 -1/CO2 1
CONV 1 CISOLID C12 1.0

BLOCK SOLP SEP
PARAM PRES=14.7
FRAC STREAM=WASC SUBSTREAM=CISOLID COMPS=C12 FRACS=1.0
FRAC STREAM=WASC SUBSTREAM=MIXED COMPS=C02 C5 FRACS=0.0 0.0
FLASH-SPECS WASC PRES=14.7 TEMP=70.0 NPHASE=1 PHASE=S
FLASH-SPECS WASCO2 PRES=14.7 TEMP=70.0 NPHASE=1 PHASE=V

BLOCK W1 CLCHNG
BLOCK W2 CLCHNG
BLOCK CHNGR8 CLCHNG
BLOCK CHNGR9 CLCHNG

; DESIGN SPECS TO SET THE CABIN ENVIRONMENT BY REGULATING THE DIFFERENT
; GAS FLOW RATES INTO THE CABIN
DESIGN-SPEC NITR
F COMMON/USRINP/B33,B11,B44,B22,NM,NW,A88
DEFINE A1 MASS-FLOW STREAM=INSAIR COMPONENT=C4
SPEC A1 TO B33
TOL-SPEC 1D-3
VARY MASS-FLOW STREAM=N2O2 COMPONENT=C4
LIMITS 1D-1 50000.0

DESIGN-SPEC WAT
DEFINE A2 MASS-FLOW STREAM=INSAIR COMPONENT=C1
SPEC A2 TO B44
TOL-SPEC 1D-3
VARY MASS-FLOW STREAM=CABH2O COMPONENT=C1
LIMITS 0 1000.0

DESIGN-SPEC OXY
DEFINE A3 MASS-FLOW STREAM=INSAIR COMPONENT=C5
SPEC A3 TO B11
TOL-SPEC 1D-3
VARY MASS-FLOW STREAM=N202 COMPONENT=C5
LIMITS 1D-1 5000.0

DESIGN-SPEC CARDI
DEFINE A4 MASS-FLOW STREAM=INSAIR COMPONENT=CO2
SPEC A4 TO B22
TOL-SPEC 1D-3
VARY MASS-FLOW STREAM=CABCO2 COMPONENT=CO2
LIMITS 0 1000.0

CONVERGENCE ALSPC NEWTON
SPEC NITR/ OXY/ CARDI/WAT
PARAM MAXIT=100

; THIS IS A FORTRAN BLOCK TO SIMULATE THE BIOLOGICAL ACTIVITIES IN A MAN
FORTRAN MAN
F COMMON/USRINP/B33,B11,B44,B22,NM,NW,A88
F DIMENSION CA1(2),VO2(2),VCO2(2),MO2(2),VE(2),Q3(2),Q2(2)
F DIMENSION Q1(2),YO2I3(2),YCO2I3(2),YN2I3(2),YH2O13(2),YO2T(2)
F DIMENSION YN2T(2),YO2M(2)
F DIMENSION YCO2M(2),YN2M(2),YH2OM(2),YO2IM(2),YN2IM(2),YCO2T(2)
F DIMENSION YCO2IM(2),YH2OIM(2),UR(2),H2OM(2),SWEAT(2)
F DIMENSION TFOOD(2),TW(2),TFAT(2),TPRO(2),TCARB(2),HU(2)
F DIMENSION CW(2),OXYG(2),HFOOD(2),DRINK(2),S(2),Q4(2)
F REAL*8 MO2
DEFINE A11 MASS-FLOW STREAM=EXHAIR COMPONENT=C5
DEFINE A22 MASS-FLOW STREAM=EXHAIR COMPONENT=CO2
DEFINE A33 MASS-FLOW STREAM=EXHAIR COMPONENT=C4
DEFINE A44 MASS-FLOW STREAM=EXHAIR COMPONENT=C1
DEFINE TRACE STREAM-VAR STREAM=TC VARIABLE=MASS-FLOW
DEFINE WRU MASS-FLOW STREAM=FD1U SUBSTREAM=CISOLID COMPONENT=C2
DEFINE CAR MASS-FLOW STREAM=SOLIDW SUBSTREAM=CISOLID COMPONENT=C12
DEFINE OXY MASS-FLOW STREAM=WAS02 COMPONENT=C5
DEFINE A55 MASS-FLOW STREAM=METW COMPONENT=C1
DEFINE A66 MASS-FLOW STREAM=SW COMPONENT=C1
DEFINE A99 MASS-FLOW STREAM=FOODW COMPONENT=C1
DEFINE A12 BLOCK-VAR BLOCK=CAB2 SENTENCE=R-FRAC & VARIABLE=FRAC IDI=A1ROUT
F OPEN(UNIT=61,FILE='INFO.DAT',STATUS='OLD')
F OPEN(UNIT=65,FILE='SPACE.OUT',STATUS='UNKNOWN')
F READ(61,*)NM,NW,AM,AW,WM,WW
; NM IS THE NUMBER OF MEN
; NW IS THE NUMBER OF WOMEN
; AM IS THE AVERAGE AGE OF MAN
; AW IS THE AVERAGE AGE OF WOMEN
; WM IS THE AVERAGE WT OF MAN
; WW IS THE AVERAGE WT OF WOMEN
; EEM IS THE BASAL MET ENERGY IN KCAL PER HR FOR MEN
; EEW IS THE BASAL METABOLIC ENERGY IN KCAL PER HR FOR WOMEN

F EEM = (WM**0.5) / (0.1015*AM**0.1333)) / 24.0
F EEW = (WW**0.5) / (0.1127*AW**0.1333)) / 24.0
F READ(61,*) ALM, ALW
; ALM AND ALW ARE NO OF TIMES THE BASAL ACTIVITY LEVEL IS THE ENERGY
; EXPENDED BY THE ACTIVITY
; LET ATM BE THE ENERGY EXPENDED BY ONE MAN IN ONE HR IN KCAL
; LET ATW BE THE ENERGY EXPENDED BY ONE WOMAN IN ONE HR IN KCAL
F ATM = EEM*ALM
F ATW = EEW*ALW
; LET RF BE THE FAT CARB RATIO
; LET RP BE THE PROTEIN CARB RATIO
F RF = 0.3
F RP = 0.35
F X = ATM
F DO 60 I = 1, 2
; CAI IS THE AMT OF CARB REQ IN GRAMS
F CAI(I) = X/(4.17+RF*9.3+RP*4.3)
; VO2 IS THE AMT OF O2 REQ IN LIT
F VO2(I) = CAI(I)*(0.85 + 0.86*RP + 2.29*RF)
; VCO2 IS THE AMT OF CO2 REQ IN LIT
F VCO2(I) = CAI(I)*(0.85 + 0.71*RP + 1.59*RF)
; YO2I IS MOLE FRACTION OF O2 REQUIRED TO BE MAINTAINED IN THE CABIN
; YN2I IS THE MOLE FRACTION OF N2 REQUIRED TO BE MAINTAINED IN THE CABIN
; YH2OI IS THE MOLE FRACTION OF WATER VAPOR TO BE MAINTAINED IN THE
; CABIN
; YCO2I IS THE MOLE FRACTION OF CO2 TO BE MAINTAINED IN THE CABIN
F YO2I = 0.208
F YN2I = 0.769
F YCO2I = 0.003
F YH2OI = 0.02
; MO2 IS THE AMT OF O2 CONSUMED IN LIT PER MIN AT STP FOR FOOD OXIDATION
F MO2(I) = (VO2(I)/60.0)*0.88
; WE MAKE USE OF A LINEAR RELATIONSHIP TO CALC AIR INTO THE LUNGS
; VE IS THE VENTILLATION RATE IN LIT PER MIN AT BTP AND SATURATED
F VE(I) = (24.55*MO2(I)) + 0.1125
; Q3 IS THE AIR INTO THE LUNGS IN LIT PER HR
F Q3(I) = VE(I) * 60.0
; LET OH BE THE AMT OF WATER PRESENT IN Q1 AS ML OF WATER PER ML OF DRY
; AIR
F OH = (YH2OI*760.0)/(760.0 - (YH2OI*760.0))
; IN THE OUTGOING GAS AMT OF WATE PRESENT PER LIT OF DRY GAS IS
; GIVEN BY ( 47/(760.0 -47) ) = 0.066
; LET Q2 BE THE VGL OF AIR ENTERING THE NASOPHARYNX IN LIT
F Q2(I) = Q3(I)/(1-OH)*(1+0.066)
; LET Q1 BE HE VOL OF AIR ENTERING THE NASAL PSG IN LIT
NOW TO FIND OUT MOLE FRACTION OF GASES ENTERING THE LUNGS

\[
\text{Y}_{O_2}(I) = \frac{Y_{O_2}I}{Q_2(I)/Q_3(I)}
\]

\[
\text{Y}_{C_0_2}(I) = \frac{Y_{C_0_2}I}{Q_2(I)/Q_3(I)}
\]

\[
\text{Y}_{N_2}(I) = \frac{Y_{N_2}I}{Q_2(I)/Q_3(I)}
\]

\[
\text{Y}_{H_2O}(I) = 0.062
\]

LET Q4 BE THE VOLUME OF AIR COMING OUT OF THE LUNGS IN LIT

\[
S(I) = Q_3(I) * (Y_{O_2}I + Y_{C_0_2}I + Y_{N_2}I)
\]

\[
Q_4(I) = \frac{S(I) - V_{O_2}(I) + V_{C_0_2}(I)}{0.938}
\]

\[
Y_{O_2}(I) = \frac{(Q_3(I) * Y_{O_2}I) - V_{O_2}(I)}{Q_4(I)}
\]

\[
Y_{C_0_2}(I) = \frac{(Q_3(I) * Y_{C_0_2}I) + V_{C_0_2}(I)}{Q_4(I)}
\]

\[
Y_{N_2}(I) = \frac{Q_3(I) * Y_{N_2}I}{Q_4(I)}
\]

TO FIND OUT AMOUNT OF O2, N2, H2O, CO2 IN EXPIRED AIR IN LB PER HR

\[
Y_{O_2}(I) = \frac{(Q_4(I) * 0.035 * Y_{O_2}I * 32.0)}{(0.7302 * 558.6)}
\]

\[
Y_{C_0_2}(I) = \frac{(Q_4(I) * 0.035 * Y_{C_0_2}I * 44.0)}{(0.7302 * 558.6)}
\]

\[
Y_{N_2}(I) = \frac{(Q_4(I) * 0.035 * Y_{N_2}I)}{(0.7302 * 558.6)}
\]

TO FIND OUT AMOUNT OF GAS IN INSPIRED AIR IN LB PER HR

\[
Y_{O_2}(I) = \frac{(Q_4(I) * 0.035 * Y_{O_2}I * 32.0)}{(0.7302 * 558.6)}
\]

\[
Y_{C_0_2}(I) = \frac{(Q_4(I) * 0.035 * Y_{C_0_2}I * 44.0)}{(0.7302 * 558.6)}
\]

\[
Y_{N_2}(I) = \frac{(Q_4(I) * 0.035 * Y_{N_2}I)}{(0.7302 * 558.6)}
\]

LET UR BE THE AMOUNT OF UREA IN URINE IN LB PER HR

\[
UR(I) = C(I) * R(I) * 0.337 * 0.0022
\]

LET H2O BE AMOUNT OF METABOLIC WATER PRODUCED IN LB PER HR

\[
H_{2O}(I) = C(I) * (0.6 + (1.125 * R(I)) + (0.51 * R(I))) * 0.0022
\]

LET SWEAT BE THE AMOUNT OF WATER LOST AS SWEAT IN LB PER HR

IF (X.LE.250.0) GO TO 41

\[
\text{SWEAT}(I) = \frac{(2.5 * X) - 325.0}{0.0022}
\]

GO TO 42

41

\[
\text{SWEAT}(I) = \frac{(1.50 * X) - 75}{0.0022}
\]

HU IS THE AMOUNT OF WATER IN URINE IN LB PER HR

\[
HU(I) = UR(I) * 69.22
\]

LET TFAT BE THE TOTAL FAT IN DIET IN GM

\[
TFAT(I) = C(I) * 1.02 * R(I)
\]

LET TPRO BE THE TOTAL PROTEIN IN DIET IN GM

\[
TPRO(I) = C(I) * 1.07 * R(I)
\]

LET TCARB BE THE TOTAL CARB IN DIET IN GM

\[
TCARB(I) = C(I) * 1.02
\]

LET TFOOD BE THE TOTAL FOOD IN DIET IN GM PER HR

\[
TFOOD(I) = TFAT(I) + TPRO(I) + TCARB(I)
\]

LET TW BE THE TOTAL WASTE IN GM PE HR

\[
TW(I) = C(I) * (0.02 + (0.02 * R(I)) + (0.07 * R(I)))
\]

LET CW BE THE TOTAL CARBON CONTENT OF WASTE IN GM PER HR

\[
CW(I) = C(I) * (0.4 * 0.02) + (0.758 * 0.02 * R(I)) + (0.312 * 0.07 * R(I))
\]

LET OXYG BE THE TOTAL O2 REQ TO BURN THE FECES IN LB PER HR

\[
OXYG(I) = CW(I) * 0.0022 \times (32.0/12.0)
\]

LET HFOOD BE AMOUNT OF WATER IN FOOD IN LB PER HR

\[
HFOOD(I) = 1.32 * TFOOD(I) * 0.0022
\]
; LET DRINK BE TOTAL AMT OF POTABLE WATER REQ IN LB PER HR
F DRINK(I)=SWEAT(I)+HU(I)+YH2OM(I)-YH2OIM(I)-HFOOD(I)-H2OM(I)
F X=ATW
F 60 CONTINUE

; CA, P, FA ARE THE TOTAL CARBOHYDRATE, PROTEIN, FAT REQUIRED BY THE CREW RESPECTIVELY (GM/HR)
F CA = TCARB(1)*NM + TCARB(2)*NW
F FA = TFAT(1)*NM + TFAT(2)*NW
F P = TPRO(1)*NM + TPRO(2)*NW
F WRITE (65,*) 'CA',CA,'FA',FA,'P',P

; A11,A22,A33,A44 ARE THE TOTAL AMOUNT OF O2,CO2,N2 AND WATER RESPECTIVELY WHICH ARE PRESENT IN THE TOTAL MASS OF EXPIRED AIR INTO THE CABIN BY ALL THE CREW MEMBERS
F A11 = YO2M(1)*NM + YO2M(2)*NW
F A22 = YCO2M(1)*NM + YCO2M(2)*NW
F A33 = YN2M(1)*NM + YN2M(2)*NW
F A44 = YH2OM(1)*NM + YH2OM(2)*NW

; B11,B22,B33,B44 ARE THE TOTAL AMOUNT OF O2,CO2,N2 AND WATER RESPECTIVELY WHICH ARE PRESENT IN THE TOTAL MASS OF INSPIRED AIR BY ALL THE CREW MEMBERS.
F B11 = YO2IM(1)*NM + YO2IM(2)*NW
F B22 = YCO2IM(1)*NM + YCO2IM(2)*NW
F B33 = YN2IM(1)*NM + YN2IM(2)*NW
F B44 = YH2OIM(1)*NM + YH2OIM(2)*NW

; LET Q6 BE THE TOTAL VOL OF INSPIRED AIR
F Q6 = QI(1)*NM + QI(2)*NW
F WRITE (65,*) 'TOTW',TOTW,'CAR',CAR

; LEP PER BE TOTAL % OF CARBON IN WASTE
F PER = CAR/(TOTW)*0.0022
F WRITE (65,*) 'PER',PER

; OXY IS THE TOTAL O2 REQUIRED FOR SOLID WASTE INCINERATOR
F OXY = OXYG(1)*NM + OXYG(2)*NW

; A55 IS TOTAL METABOLIC WATER PRODUCED BY ALL THE CREW MEMBERS
F A55 = H2OM(1)*NM + H2OM(2)*NW

; A66 IS THE TOTAL SWEAT GENERATED BY ALL THE CREW MEMBERS
F A66 = SWEAT(1)*NM + SWEAT(2)*NW

; A77 IS THE TOTAL AMOUNT OF WATER ASSOCIATED WITH THE UREA AS URINE
F A77 = HU(1)*NM + HU(2)*NW

; A88 IS THE TOTAL POTABLE WATER TO BE SUPPLIED AS DRINKING WATER
F A88 = DRINK(1)*NM + DRINK(2)*NW

; A99 IS THE TOTAL WATER ASSOCIATED IN THE FOOD REQUIRED BY THE ENTIRE CREW
F A99 = HFOOD(1)*NM + HFOOD(2)*NW
; LET THEAT BE THE TOTAL HEAT LIBERATED BY THE CREW IN BTU
F THEAT = (NM*ATM + NW*ATW) *(4187.0/1055.0)
; LET THE BUFFER TEMP OF THE GASES SUPPLIED TO THE CABIN BE 69 F
; LET Q5 BE THE VOL OF AIR TO BE SUPPLIED
F Q5 = THEAT/(1.08*1.0)
F A10 = (Q5*60.0)/(Q6*0.035)
F A12 = A10 / (1+A10)
F WRITE(65,*) 'A99',A99,'A88',A88,'AI2',A12
WRITE-VARS A11 A22 A44 A33 TRACE WRU CAR OXY A55 A99 A66 & A12
FLASH-SPECS EXHAIR TP NPHASE=1
FLASH-SPECS METW NOFLASH
FLASH-SPECS FOODW NOFLASH
; THIS IS A FORTRAN BLOCK TO REGULATE THE WATER REQUIRED FOR SATURATING
; THE EXPIRED AIR
FORTRAN WSEPAN
DEFINE WSEP1 MASS-FLOW STREAM=INSAIR COMPONENT=C1
DEFINE WSEP2 MASS-FLOW STREAM=EXHAIR COMPONENT=C1
DEFINE WSEP3 BLOCK-VAR BLOCK=WASEP SENTENCE=MASS-FLOW & VARIABLE=FLOW IDI=WAEXH
DEFINE WSEP4 MASS-FLOW STREAM=SW COMPONENT=C1
DEFINE WSEP5 BLOCK-VAR BLOCK=WASEP SENTENCE=MASS-FLOW & VARIABLE=FLOW IDI=SW1
F WSEP3 = WSEP2 - WSEP1
F WSEP5 = WSEP4
EXECUTE BEFORE WASEP
; THIS DESIGN SPEC IS USED TO REGULATE THE AMOUNT OF CO2 REMOVED BY
; ENSURING THAT AMOUNT OF CO2 IN THE AIR STREAM IS WHAT IS DESIRED TO BE
; SUPPLIED TO THE CABIN. HERE WE MAKE AN ASSUMPTION THAT THE EQUIPMENT
; IS AVAILABLE TO ACHIEVE THE DESIRED SEPARATION.
DESIGN-SPEC SEPATR
DEFINE SEP1 MASS-FLOW STREAM=ABAIR1 COMPONENT=C02
DEFINE SEP2 MASS-FLOW STREAM=CABC02 COMPONENT=C02
SPEC SEP1 TO SEP2
TOL-SPEC ID-3
VARY BLOCK-VAR BLOCK=SEP3 SENTENCE=FRAC & VARIABLE=FRACS IDI=MIXED ID2=ABAIR1 ELEMENT=1
LIMITS 0 1.0
; THIS FORTRAN BLOCK IS USED TO ADJUST THE AMOUNT OF H2 TO BE SUPPLIED
; TO THE BOSCH PROCESS.
FORTRAN H2ADJUST
DEFINE HY1 MASS-FLOW STREAM=HYDRO COMPONENT=C3
DEFINE HY2 MASS-FLOW STREAM=SEPOUT2 COMPONENT=C3
DEFINE HY3 STREAM-VAR STREAM=HYDRO VARIABLE=VFRAC
DEFINE CO1 STREAM-VAR STREAM=SEPC VARIABLE=MASS-FLOW
DEFINE CO4 STREAM-VAR STREAM=WASCO21 VARIABLE=MASS-FLOW
DEFINE CO5 MASS-FLOW STREAM=SEPOUT2 COMPONENT=C02
F HY1 = 2*(CO1+CO5+CO4) - HY2
F HY3 = 1.0
F WRITE(65,*) 'HY1',HY1,'HY3',HY3
READ-VARS HY2 CO1 CO5 CO4
WRITE-VARS HY1 HY3

; THIS FORTRAN BLOCK IS USED TO CONTROL THE TEMPERATURE OF THE
; DEHUMIDIFIER FOR THE EXIT AIR FROM THE CABIN.

FORTRAN HUMIDIFICATION

DEFINE HU1 MASS-FLOW STREAM=HUHXOUT COMPONENT=C1
DEFINE HU2 MASS-FLOW STREAM=CAH2O COMPONENT=C1
DEFINE HU2A BLOCK-VAR BLOCK=HEA2 SENTENCE=FRAC &
VARIABLE=FRACS IDImMIXED ID2=CAIR ELEMENTml
DEFINE HU3 STREAM-VAR STREAM=HUHXOUT VARIABLE=MASS-FLOW
DEFINE HTEMP STREAM-VAR STREAM=HUHXOUT VARIABLE=TEMP
DEFINE HTSEP1 BLOCK-VAR BLOCK=HEA2 SENTENCE=FLASH-SPECS &
VARIABLE=TEMP ID1=WA
DEFINE HTSEP2 BLOCK-VAR BLOCK=HEA2 SENTENCE=FLASH-SPECS &
VARIABLE=TEMP ID1=CAIR

F HU4 = HU3 - HU1
F HU2A = HU2/HU1
F HYIN = HU1/HU4
F HYOUT = HU2/HU4
F LABAS = 1075.8
F CS = 0.24 + 0.45*HYIN
F HU5 = ((HYIN - HYOUT)*LABAS)/CS
F HTSEP1 = HTEMP - HU5
F HTSEP2 = HTEMP - HU5
F WRITE(65,*)'HTSEP2',HTSEP2,'HU5',HU5
READ-VARS HU1 HU2 HU3 HTEMP
WRITE-VARS HTSEP1 HTSEP2 HU2A

; THIS DESIGN SPEC IS USED TO REGULATE THE AMOUNT OF WATER TO BE
; SUPPLIED FOR DRINKING. THIS ENABLES TO COMPLETELY CLOSE THE WATER
; LOOP. THE REMAINING WATER IS SENT TO THE WATER ELECTROLYSIS UNIT.

DESIGN-SPEC MAKEUPWATER

DEFINE AK1 MASS-FLOW STREAM=MAKEUPWATER COMPONENT=C1
DEFINE AK2 MASS-FLOW STREAM=DECAN COMPONENT=C1
DEFINE AK3 MASS-FLOW STREAM=WAI COMPONENT=C1
F AK5 = A88 - (AK2+AK3)
F IF(AK5.LE.0.0) GO TO 71
F AK6 = AK5
F GO TO 72
F 71 AK6 = 0.0
F 72 WRITE(65,*)'AK6',AK6
SPEC AK1 TO AK6
TOL-SPEC ID=3
VARY BLOCK-VAR BLOCK=SPTT4 SENTENCE=FRAC &
VARIABLE=FRAC ID1=MAKEUPWATER
LIMITS 0 1.0

; THIS FORTRAN BLOCK IS USED TO COMPUTE THE AMOUNT OF MAKEUP HYGIENE
; WATER REQUIRED

FORTRAN HYGIENEMAKEUP

DEFINE UI1 MASS-FLOW STREAM=S2 COMPONENT=C1
DEFINE UI2 MASS-FLOW STREAM=MAKEUPS2 COMPONENT=C1
OPEN(UNIT=90,FILE='WA.OUT',STATUS='UNKNOWN')
LET UI3 BE THE FLOW RATE OF MAKEUP HYGIENE WATER
126

F U13 = 2.29 * (NM+NW) - U11
F IF (U13.LT.0.0) GO TO 311
F U12 = U13
F GO TO 312
F 311 U12 = 0.0
F 312 WRITE (65, *) U12, 'AMT OF MAKUP HYGIENE WATER'
EXECUTE BEFORE MIX18
FLASH-SPECS S2 NOFLASH
FLASH-SPECS MAKEUPS2 NOFLASH

; THIS IS A FORTRAN BLOCK TO MAINTAIN THE UREA CONC AS DESIRED IN THE
; THE HYGIENE WATER
FORTRAN URMAINT
   DEFINE R1 MASS-FLOW STREAM=S21 SUBSTREAM=CISOLID COMPONENT=C2
   DEFINE R2 STREAM-VAR STREAM=S2 SUBSTREAM=CISOLID VARIABLE=MASS-FLOW
   OPEN (UNIT=90, FILE='WA.OUT', STATUS='UNKNOWN')
   R22 = 5.5D-5 * (NM+NW) - R1
   IF (R22.LT.0.0) GO TO 410
   R2 = R22
   GO TO 411
F 410 R2 = 0.0
F 411 WRITE (65, *) R2, 'MASS FLOW OF SS2'
EXECUTE BEFORE MIX15
FLASH-SPECS S2 NOFLASH
FLASH-SPECS SS2 NOFLASH

; THIS IS A FORTRAN BLOCK TO COMPUTE THE MAKEUP URINAL FLUSH WATER
; REQUIRED
FORTRAN FLUSHMAK
   DEFINE FL1 MASS-FLOW STREAM=S1 COMPONENT=C1
   DEFINE FL2 MASS-FLOW STREAM=URFLUSH COMPONENT=CI
   FL3 = (0.0454 * (NM+NW)) - FL1
   IF (FL3.LT.0.0) GO TO 99
   FL2 = FL3
   GO TO 97
F 99 FL2 = 0.0
F 97 WRITE (65, *) 'URFUSH REQ', FL2
EXECUTE BEFORE MIX20
FLASH-SPECS S1 NOFLASH

; THIS FORTRAN BLOCK IS USED TO DETERMINE THE SPLIT FRACTION FOR THE
; PURE OBTAINED FROM THE WATER REGENERATION UNIT AS IN SPTT3
FORTRAN WATERSPLIT
   DEFINE WA1 STREAM-VAR STREAM=CON3 VARIABLE=MASS-FLOW
   DEFINE WA2 BLOCK-VAR BLOCK=SPTT3 SENTENCE=MASS-FLOW &
      VARIABLE=FLOW ID1=S1
   DEFINE WA3 BLOCK-VAR BLOCK=SPTT3 SENTENCE=MASS-FLOW &
      VARIABLE=FLOW ID1=S2
; LET HY BE THE AMOUNT OF WATER REQ FOR HYGIENE PURPOSE
; LET FLU BE THE AMOUNT OF WATER REQ FOR FLUSH PURPOSE
; THESE ABOVE VALUES ARE OBTAINED FROM NASA DATA BASE FOR A CREW OF ONE
; MEMBER
F HY = 2.29 * (NM+NW)
F FLU = 0.0454 * (NM+NW)
; LET WA5 BE THE DIFF BETWEEN CON3 AND FLU AND HY
127

F  WA5=W1 - (HY + FLU)
F  IF(WA5.LT.0.0)GO TO 13
F  WA2 = FLU
F  WA3 = HY
F  GO TO 133
F  13 WA2=0.02*WA1
F  WA3=0.98*WA1
F  133 WRITE(65,*)'WA2',WA2,'WA3',WA3
EXECUTE BEFORE SPTT3
FLASH-SPEC CON3 NOFLASH

; THIS FORTRAN BLOCK IS USED TO COMPUTE THE SPLIT FRACTION FOR THE
; CATALYTIC OXIDATION IN THE TRACE GAS TREATMENT SUBSYSTEM
FORTRAN TRACRAT
DEFINE TR1 MASS-FLOW STREAM=OUTM COMPONENT=C6
DEFINE TR2 STREAM-VAR STREAM=OUTM VARIABLE=MASS-FLOW
DEFINE TR3 BLOCK-VAR BLOCK=SPLTI SENTENCE=FRAC &
VARIABLE=FRAC IDI=HT
F  TR4 = TR1/TR2
F  TR5 = (0.075/0.08)*TR4
F  IF (TR5.LE.0.075)GO TO 92
F  TR7 = TR5
F  GO TO 93
F  92 TR7 = 0.075
F  93 IF (TR7.GT.0.3)GO TO 95
F  TR3=TR7
F  GO TO 96
F  96 WRITE(65,*)'TR3',TR3
EXECUTE BEFORE SPLTI

; THIS FORTRAN BLOCK IS USED TO COMPUTE THE AMOUNT OF EXPENDABLES
; STORED/CONSUMED FOR THE ENTIRE LSS
FORTRAN REQUIREMENTS
DEFINE RHQ2 MASS-FLOW STREAM=HYD1 COMPONENT=C3
DEFINE RHQ3 MASS-FLOW STREAM=HYDRO COMPONENT=C3
DEFINE RCQ4 MASS-FLOW STREAM=ABAIR COMPONENT=CO2
DEFINE RCQ5 MASS-FLOW STREAM=CABCO2 COMPONENT=C02
DEFINE RNQ6 MASS-FLOW STREAM=ABAIR COMPONENT=C4
DEFINE RNQ7 MASS-FLOW STREAM=N2O2 COMPONENT=C4
DEFINE ROQ8 MASS-FLOW STREAM=N2O2 COMPONENT=C5
DEFINE ROQ9 MASS-FLOW STREAM=ABAIR COMPONENT=C5
DEFINE ROQ10 MASS-FLOW STREAM=OXYG1 COMPONENT=C5
DEFINE RWQ12 MASS-FLOW STREAM=CABH2O COMPONENT=C1
DEFINE RWQ13 MASS-FLOW STREAM=WASO2 COMPONENT=C5
DEFINE RWQ14 MASS-FLOW STREAM=MAKEUPS2 COMPONENT=C1
DEFINE RWQ15 MASS-FLOW STREAM=URFLUSH COMPONENT=C1
DEFINE RWQ16 MASS-FLOW STREAM=ABAIR COMPONENT=C1
DEFINE RWQ17 STREAM-VAR STREAM=SOLIDC SUBSTREAM=MIXED &
VARIABLE=MASS-FLOW
F  OPEN(UNIT=66,FILE='REPORT.OUT',STATUS='UNKNOWN')
; H2REQ IS AMT OF H2 IN BUFFER
F  H2REQ = RHQ2 - RHQ3
F  CO2REQ = RCQ5 - RCQ4
F  N2REQ = RNQ7 - RNQ6
; O2REQ IS AMT OF O2 STORED IN BUFFER
F O2TCAB = ROQ8 - ROQ9
F O2REQ = ROQ10 + ROQ9 - ROQ13 - ROQ8
F WATREQ = RWQ12 + RWQ14 + RWQ15 - RWQ16
F WRITE(66,*) 'NM:', NM, 'NW:', NW
F WRITE(66,*) 'AMOUNT OF WATER REQ:', WATREQ
F WRITE(66,*) 'AMOUNT OF OXY REQ:', O2REQ
F WRITE(66,*) 'AMOUNT OF CO2 REQ:', CO2REQ, 'AMT OF N2 REQ:', N2REQ
F WRITE(66,*) 'AMOUNT OF H2', H2REQ
F WRITE(66,*) 'AMOUNT OF SOLIDC', RWQ17
F READ-VARS RHQ2 RHQ3 RCQ4 RCQ5 RNQ6 RNQ7 ROQ8 ROQ9 ROQ10 &
  RWQ12 RWQ13 RWQ14 RWQ15 RWQ17

; THIS IS A FORTRAN BLOCK TO DETERMINE THE WASTE OXIDIZER TEMP (A LINEAR
; RELATIONSHIP IS USED)
FORTTRAN WASOTEMP
  DEFINE CTEMP BLOCK-VAR BLOCK=SOLIDINC SENTENCE=PARAM &
    VARIABLE=TEMP
  F CTEMP = (((194 - PER)/0.65)*(9/5)) + 32
  F WRITE(65,*) 'CTEMP', CTEMP, 'PER', PER
  WRITE-VARS CTEMP

; THIS IS A FORTRAN BLOCK TO DETERMINE THE AMOUNT OF WATER IN BRINE.
; THIS IS TO ENSURE THAT THE OUTPUT FROM THE VCD IS A 45% CONCENTRATED
; SOLUTION
FORTTRAN VCD
  DEFINE VCD1 MASS-FLOW STREAM=FOUT SUBSTREAM=CISOLID COMPONENT=C2
  DEFINE VCD2 MASS-FLOW STREAM=FOUT COMPONENT=C1
  DEFINE VCD3 BLOCK-VAR BLOCK=STILL SENTENCE=FRAC &
    VARIABLE=FRACS IDI=MIXED ID2=BRINE ELEMENT=1
  F VCD4 = 0.45*VCD1
  F VCD3 = VCD4/VCD2
  F WRITE(65,*) 'VCD3', VCD3
  READ-VARS VCD1 VCD2
  WRITE-VARS VCD3

; THIS IS A FORTRAN BLOCK TO DOUBLY ENSURE THAT THE RIGHT AMOUNT OF
; POTABLE WATER IS SUPPLIED TO THE CABIN. THE EXCESS IS SENT FOR O2
; GENERATION
FORTTRAN POTAD
  DEFINE POT1 BLOCK-VAR BLOCK=POTSEP SENTENCE=MASS-FLOW &
    VARIABLE=FLOW IDI=PWO1
  F POT1 = A88
  F WRITE(65,*) 'POT1', POT1
  EXECUTE AFTER POTMIX

CONV-OPTIONS
  PARAM TEAR-METHOD=WEGSTEIN TOL=0.0001
  WEGSTEIN MAXIT=200

RUN-CONTROL MAX-TIME=4000

BLOCK-REPORT NEWPAGE
APPENDIX 2
Body Water Compartments

The most abundant compound in the human body is water, accounting for 40% to 80% of the body weight depending upon the amount of adipose tissue present. The body contains the highest proportion of water at infancy, up to 80%, and then the water content decreases as old age is approached. The average water content of an adult male is 60% of the body weight whereas the average water content of an adult female is 50% of the body weight. This difference is due to the content of subcutaneous fat.

The total body water can be categorized as existing in two main compartments: intracellular water and extracellular water. The intracellular water consists of all the water within the cells and constitutes over half of the total body water. Since red blood cells are surrounded by plasma, and all other cells are surrounded by interstitial fluid, the intracellular compartment has been sub-divided to represent these two cell types in figure 1. The extracellular water, which includes all of the fluid outside of the cells, can be further subdivided into compartments which represent interstitial fluid, circulating blood plasma, lymph, and transcellular water. The interstitial fluid surrounds cells outside of the vascular system whereas plasma is contained within the blood vessels. Avascular tissues such as dense connective tissue and cartilage contain interstitial water which slowly equilibrates with tracers used to determine extracellular fluid volume. For this reason, additional compartments are sometimes used to represent these avascular tissues. Lymph is interstitial fluid which has flowed into the lymphatic vessels and is eventually returned to the vascular system. A small fraction of the extracellular fluid is separated from other extracellular compartments by a layer of epithelium. This compartment, which includes cerebrospinal fluid, aqueous and vitreous humor, synovial fluid, and fluid secretions of glands, makes up the transcellular compartment. These body water compartments, and the interactions between them, are shown in figure 1. The average size of each compartment, in terms of percent body weight, has been determined for adult males and females (West, 1985) and is included in this figure. The lymph compartment, which is often included in the interstitial compartment, has been determined to be approximately 2.0% of the body weight (Pitts, 1974). The size of the lymph compartment for females was determined by multiplying 2.0% by 5/6 (recall that the average female body is 50% water and the average male body is 60% water).

Cerebrospinal fluid and aqueous humor, which make up a part of the transcellular compartment, are formed by epithelial cells secreting sodium ions into the associated chamber (Guyton, Taylor, & Granger, 1975). The secretion of positively charged sodium ions creates a potential difference, with respect to the blood, thereby causing negatively charged ions to flow from the plasma, into the epithelial cells, and eventually into the chambers. The excess ions then cause water to flow across the epithelial membranes by osmosis. Therefore, water flows from the intracellular fluid of the epithelial cells into the chambers by osmosis. Since osmosis is also the primary force for the flow of water between the cellular compartment and the interstitial compartment, and since
the transcellular compartment accounts for less than 2% of the body weight, this compartment has been lumped together with the interstitial compartment in figure 1.

**Osmosis**

Osmosis refers to the movement of solvent across a semipermeable membrane which is permeable to solvent but not to solutes. A cell membrane, which is freely permeable to water but not to most solutes, is an example of such a semipermeable membrane. Osmosis is similar to diffusion in that a concentration gradient is the driving force behind the movement of molecules, however, diffusion refers to the movement of solute molecules whereas osmosis refers to the movement of solvent molecules. Additionally, osmosis and diffusion occur in opposite directions since when the concentration of solute molecules is high, the concentration of water must be low. The process of osmosis is illustrated in figure 2, where the small spheres represent water molecules and the large spheres represent solute molecules. The two solutions on each side of the semipermeable membrane initially have different solute concentrations (step 1). The solvent molecules, in this case water, begin crossing the membrane from the side with the lower solute concentration to the side with the higher solute concentration. This process continues until the concentration solute is the same on both sides (step 2a). The larger the solute concentration difference on each side, the stronger the driving force, or osmotic force, for water movement. This osmotic force is sometimes called the osmotic pressure. If a downward pressure is applied to side b (step 2b), the movement of water across the membrane can be eliminated and this downward pressure is equal to the osmotic pressure.

![Figure 2: An example of osmosis](image)

Figure 2: An example of osmosis where the small spheres represent water molecules and the large spheres represent solute molecules. (taken from Martini, 1989)
Females where BW = body weight, M = male, F = female.

The size of each compartment is shown for both males and females with the size of each compartment represented by K. The size of each compartment is shown for both males and females with the size of each compartment represented by K.

Figure 1: Body water compartments and the forces which cause flow between them. Mass transfer processes include osmosis, absorption, and processes.
Interactions between Body Water Compartments

Water enters the body through the digestive tract and moves across the mucosa of the small and large intestines in response to osmotic gradients (Ganong, 1991). Previous studies have estimated that water is absorbed from the gastrointestinal tract (GI tract) at a constant rate with a zeroth order rate constant of 3.3 hr\(^{-1}\) (Reeve & Guyton, 1967). The volume of the stomach, which is given in figure 1, is about one liter (Hole, 1987). Water is also produced by the intracellular metabolism of nutrients. The movement of water between the fluid compartments is controlled by hydrostatic pressure, osmotic pressure, or both.

Fluid movement between the extracellular and intracellular compartments is built on three main points: (1) that water can easily move between the two compartments; (2) that most of the solutes on each side of the cell membrane will not penetrate the membrane easily; and (3) that hydrostatic pressure differences do not play a major role in the final fluid distribution (Coleman, Norman, & Manning in Guyton, Taylor, & Granger, 1975). In other words, water will cross the cell membrane until osmotic equilibrium has been attained between the two compartments. Hydrostatic pressures are not a major factor in this osmotic equilibrium since the cell wall is extremely flexible and marked volume changes do not produce significant intracellular hydrostatic pressure (Coleman, Norman, & Manning in Guyton, Taylor, & Granger, 1975).

To determine the rate at which water penetrates the cells, the cell membrane permeability and surface area must be known. The hydraulic permeability of the human cell membrane to water is approximately 3.0 μ\(^3\) water/μ\(^2\)·atm (West, 1985). The surface area of the red blood cell compartment, which is in osmotic equilibrium with the plasma compartment, can be calculated from the blood volume, hematocrit (Hct), red blood cell volume, and red blood cell surface area. The blood volume can be calculated with the following equation.

\[
\text{Blood volume} = \frac{\text{Plasma volume}}{1 - \text{Hct}}
\]

The average normal hematocrit is 0.47 for men and 0.42 for women (Ganong, 1991). The volume and surface area of a normal red blood cell are 9.7x10\(^{-8}\) μl and 135 μm\(^2\), respectively (West, 1985). The surface area of the non-red blood cell compartment, which is in osmotic equilibrium with the interstitial fluid, can be calculated from the average cell size and the volume of the non-red blood cell compartment. The average cell is roughly cubic with dimensions of 10μm x 10μm x 10μm (Martini, 1989).

The exchange of fluid between the plasma and interstitial compartments can be described by what are called Starling forces. This exchange occurs
across the capillary walls and is responsible for supplying cells with oxygen and nutrients while removing cellular wastes. Capillary hydrostatic pressure and interstitial osmotic pressure forces fluid out of the capillaries and into the interstitial spaces. Conversely, the interstitial hydrostatic pressure and the plasma osmotic pressure force fluid back into the capillaries. The composition of the plasma is almost identical to that of the interstitial fluid with the exception of the protein content. Most of the relatively large protein molecules cannot penetrate the capillary wall, therefore, the protein content is substantially higher in the plasma and causes the osmotic effects. The protein osmotic pressure is sometimes referred to as the oncotic pressure. Typically the plasma hydrostatic pressure declines from about 37 mm Hg to 17 mm Hg along the length of the capillary whereas the oncotic pressure and interstitial hydrostatic pressure remain relatively constant at 25 mm Hg and 1 mm Hg, respectively. Consequently, fluid is forced out of the arteriole end of the capillary and fluid is forced into the venous end of the capillary. This situation is shown in figure 3.

![Figure 3: Representation of pressure gradients across the wall of a muscle capillary. The arrows indicate the approximate magnitude and direction of fluid movement. (taken from Ganong, 1991)](image)

Fluid movement is related to the Starling forces through the following expression.

\[
\text{Fluid movement} = K_f[(P_c + \pi_i) - (P_i + \pi_c)]
\]

(2)

where \(K_f\) = capillary filtration coefficient  
\(P_c\) = capillary hydrostatic pressure  
\(P_i\) = interstitial hydrostatic pressure  
\(\pi_c\) = capillary oncotic pressure  
\(\pi_i\) = interstitial oncotic pressure

For the entire body, \(K_f\) has been found to be approximately 0.061 ml fluid/min·kg body weight·mm Hg (Landis & Pappenheimer, 1963).
During a 24 hour period, about 2 liters more fluid is filtered across the capillary walls than is reabsorbed (Little, 1989). This fluid then flows into the lymphatic system due to a hydrostatic pressure difference. However, since the flow rate of this interstitial fluid and the distance to a lymphatic capillary is extremely small, the pressure gradient between the interstitial spaces and lymphatic capillary is too slight to be measurable (Guyton, Taylor, & Granger, 1975). A primary characteristic of the lymphatic system is that under normal conditions any excess fluid that collects in the tissues is returned back to the circulation (Guyton, 1984). Previous studies have found that the rate of lymph flow can increase up to 20 times the resting level (Guyton, Taylor, & Granger, 1975). This phenomena is mainly due to the structure of the lymphatic capillaries. Endothelial cells of the lymphatic vessels overlap to form pores (see figure 4). As the interstitial space fills with liquid, the tissue swells and the endothelial cells are pulled apart causing the pores to open wider. Therefore, the greater the tissue pressure, the greater the lymph formation rate. The overlapping edges of the endothelial cells also prevent fluid from flowing out of the lymphatic capillary so any compressive force will cause lymph to flow forward through the vessel (Guyton, 1984).

![Figure 4: Structure of the lymphatic capillaries which allows for the variable, one directional flow of lymph. (taken from Guyton, 1984)](image_url)

For the above stated reasons, it is assumed that the lymph flow rate is equal to the net formation of interstitial fluid in figure 1. Figure 5 illustrates the effect of capillary pressure on interstitial fluid pressure, interstitial fluid volume, and lymph flow. From figure 5, it can be seen that the interstitial fluid volume remains relatively constant as the capillary pressure and interstitial pressure are increased until edema, which is an accumulation of fluid in the tissue spaces, occurs. Over this range of capillary pressures, the lymph flow
increases 15 to 20 times normal. Therefore, the lymphatic regulatory system
which prevents the buildup of interstitial fluid performs close to its maximum
level before edema occurs. Once edema results, the lymphatic system
continues to operate at the maximum level thereby alleviating the swelling as
quickly as possible.

Figure 5: Computed effects of progressive increase in capillary pressure on
interstitial fluid pressure, interstitial fluid volume, and lymph flow.
(taken from Guyton, Taylor, and Granger, 1975)

Regulation of Body Water

Water can enter the body as a liquid, with moist food, and as the result of
intracellular oxidative metabolism of various nutrients. The primary regulator
of water intake is thirst, and although the thirst mechanism is poorly
understood, it seems to involve the osmotic pressure of extracellular fluid and a
thirst center in the hypothalamus (Hole, 1987).

Water is lost from the body through four routes: sensible perspiration,
insensible perspiration, urine, and feces. Water lost by sensible perspiration, or
sweating, is a necessary part of the body's temperature control mechanism;
water loss in feces accompanies the elimination of undigested food materials;
and water losses through diffusion and evaporation (i.e. insensible perspiration)
are largely unavoidable (Hole, 1987). Consequently, the only significant route of
water loss which can be regulated is the formation of urine. The urine volume,
which can vary from less than 1 liter/day to more than 20 liters/day, is
determined primarily by the blood pressure and the plasma level of antidiuretic
hormone (ADH) (West, 1985). ADH is secreted from the posterior pituitary.
gland and the secretion rate is controlled by a center located in the hypothalamus.

The osmolality of the body fluids is regulated by thirst and the renal excretion rate of electrolytes. Ingesting hypotonic liquids will dilute the extracellular water compartment and therefore reduce plasma osmolality. ADH and another hormone, aldosterone, regulate the urinary loss of sodium and potassium.

The thirst center and the ADH secretion rate are stimulated mainly by two physiological conditions, increases in plasma osmolality and decreases in plasma volume (West, 1985). Changes in plasma osmolality are sensed by nerve cells called osmoreceptors located in the hypothalamus. It is believed that the osmoreceptors become reduced in volume by osmotic dehydration when plasma osmolality is elevated, which triggers thirst and ADH release (Cowley in Guyton, Taylor, & Granger, 1975). Changes in plasma volume are sensed by stretch receptors located in the heart and blood vessels and by the juxtaglomerular apparatus of the kidney. The stretch receptors are stimulated by distention and their afferent nerve fibers pass via the glossopharyngeal and vagus nerves to the medulla (Ganong, 1991). The juxtaglomerular apparatus secretes an enzyme, renin, in response to a decreased blood volume or decreased blood pressure. Renin converts a circulating blood protein which is produced by the liver, angiotensinogen, to angiotensin I. Angiotensin I is then converted in the lungs to angiotensin II. The actions of angiotensin II include vasoconstriction, which elevates the blood pressure, increased secretion rates of aldosterone and ADH, and stimulation of the thirst center. A change in plasma osmolality of 1% doubles the plasma level of ADH, and the thirst center is stimulated when the plasma osmolality changes 1 to 2%, whereas the change in blood volume necessary to cause these responses is on the order of 10 to 15% (Cowley in Guyton, Taylor, and Granger, 1975). However, if the osmoreceptors and volume receptors provide conflicting information, for example a low blood volume with a low osmolality, the volume regulating mechanism will be the only observed mechanism. In other words, volume overrides tonicity. Since the stretch receptors adapt to abnormal blood volumes over a period of several days, and the osmoreceptors do not adapt to abnormal plasma osmolalities (Cowley in Guyton, Taylor, and Granger, 1975), the volume regulating mechanism is valid only when considering short term regulation. The osmolality mechanism will be valid for both short term and long term body water regulation. The effect of plasma osmolality and volume on the circulating levels of ADH are shown in figure 6. The relationship between the intensity of thirst and the plasma osmolality is shown in figure 7.

An increase in arterial pressure increases the amount of fluid the kidneys withdraw from the vascular compartment and hence increases the urine flow rate. The relationship between renal arterial pressure and urine production is shown in figure 14 and will be discussed in the following section.
Description of the kidney

The urinary system includes the kidneys, ureters, bladder, and urethra. The function of this system is to maintain homeostasis of the body fluids by adjusting the composition of circulating blood. The kidneys receive about 25%

![Graph](image)

Figure 6: Control of plasma ADH concentration by osmolality, mean arterial pressure, and circulatory volume. (taken from West, 1985)

![Graph](image)

Figure 7: Relation of plasma osmolality to thirst in healthy adult humans during infusion of hypertonic saline. The intensity of thirst is measured on a special analog scale. (taken from Ganong, 1991)
of the cardiac output and filters out a fluid similar to plasma. The composition of this filtered fluid changes as it flows through the kidney tubules since compounds are continually being secreted and reabsorbed. Ultimately, the plasma-like fluid becomes urine. Through this mechanism, the kidneys eliminate wastes while conserving body water, electrolytes, and metabolites.

The kidneys are shaped similar to lima beans and weigh about 300 grams apiece. The internal structure of the kidney can be divided into two parts, an outer portion called the cortex and an inner portion called the medulla (see figure 8). The medulla of each kidney contains 6-18 conical renal pyramids, whose tips, or papillae, are each surrounded by a minor calyx (Martini, 1989). Several minor calyces combine to form a major calyx and the major calyces join within the renal pelvis which is connected to a ureter.

![Diagram of kidney anatomy](image)

Figure 8: Gross anatomy of the right kidney, in sectional view. (taken from Martini, 1989)

The nephron is the functional unit of the kidney and consists of a renal tubule and an expanded end or Bowman’s capsule (see figure 9). Each human kidney contains about 1.3 million nephrons. The Bowman’s capsule surrounds a capillary bed, the glomerulus, which receives blood through an afferent arteriole and discharges blood, less some filtrate, through an efferent arteriole. Fluid must penetrate three layers, the fenestrated capillary endothelium, the basement membrane which surrounds the capillary wall, and the glomerular epithelium, before it can enter the capsular space (see figure 10). The structure of these membranes allow water and other small molecules to cross easily but restricts the passage of larger molecules such as plasma proteins. The forces which control the filtration of plasma fluid into the Bowman’s capsule include the capillary hydrostatic pressure and capsular oncotic pressure, which force
Figure 9: The functional nephron. (taken from Martini, 1989)

Figure 10: Functional structure of the glomerular membrane. (taken from Guyton, 1984)
fluid into the capsular space, and the plasma oncotic pressure and capsular hydrostatic pressure, which force fluid into the capillaries. The renal capillary pressure is normally 60 mm Hg, whereas the colloid pressure in the glomerulus is normally 32 mm Hg (Guyton, 1984). The pressure in Bowman's capsule is about 18 mm Hg, and the colloid osmotic pressure in the capsule is essentially zero (Guyton, 1984). The capillary hydrostatic pressure may be varied considerably by the constriction of the afferent or efferent arterioles. The rate at which filtrate enters the capsule is called the glomerular filtration rate (GFR). Typically, 180 liters of fluid enter the renal tubules each day although only about one liter of urine is produced due to the secretion and reabsorption processes which occur along the length of the nephron.

The glomerular filtrate passes from the capsular space into the proximal convoluted tubule due to a hydrostatic pressure gradient. Simple cuboidal epithelial cells with microvilli line this portion of the nephron. The function of the proximal tubule is to actively reabsorb electrolytes and nutrients from the filtered fluid. As these solutes are absorbed, water flows into the epithelial cells and eventually into the interstitial fluid by osmosis. Consequently, the tubular fluid remains isotonic with respect to plasma as it travels through the proximal tubule. By the time the tubular fluid reaches the next segment of the nephron, the loop of Henle, 60 - 70% of the filtered solute and water have been removed (Ganong, 1991).

From figure 9, it can be seen that the loop of Henle consists of a descending limb and an ascending limb, each of which have thin and thick segments. The length of the loop Henle depends upon the location of the nephron within the kidney. Nephrons in the outer portions of the kidney, or cortical nephrons, have short loops whereas nephrons closer to the medulla, or juxtamedullary nephrons, have longer loops. For the juxtamedullary nephrons, a concentration gradient exists within the interstitial fluid along the length of the loop of Henle. The variation of osmolality within the medulla is shown in figure 11.

The descending limb of Henle, which is composed mainly of the thin segment, is freely permeable to water and relatively impermeable to ions (Martini, 1989). Since the osmolality of the interstitial fluid increases with depth into the medulla, water is reabsorbed from this segment by osmosis. The ascending thin limb of Henle is impermeable to water and permeable to sodium. The ascending thick limb is also impermeable to water and actively reabsorbs sodium. Since the rate of active transport is proportional to the concentration, more sodium is reabsorbed in the deeper portion of the ascending limb than in the superficial portion of the ascending limb. This active transport aids in maintaining the osmotic gradient within the medullary interstitial fluid. By the time the tubular fluid reaches the next portion of the nephron, the distal convoluted tubule, the osmolality has fallen to 100 mosm.
Near the portion of the nephron where the afferent and efferent arterioles permeate the Bowman's capsule, the ascending limb of Henle ends and forms a tight bend that places a portion of the distal tubule in direct contact with the arterioles (see figure 12). The cells of the distal tubule which contact the arterioles are known as the macula densa, and the associated smooth muscle cells in the wall of the afferent arteriole are called the juxtaglomerular cells. Together, the macula densa and the juxtaglomerular cells make up the juxtaglomerular apparatus, a secretory complex which releases renin and erythropoietin in response to a lowered blood pressure. Renin converts circulating angiotensinogen to angiotensin I which is converted to angiotensin II, a powerful vasoconstrictor, in the lung capillaries. Angiotension II also increases the secretion rates of ADH and aldosterone. The renin-angiotensin system is shown in figure 13. Erythropoietin stimulates the formation of red blood cells in the bone marrow and therefore maintains or in some cases increases the oxygen carrying capacity of the blood.

The distal convoluted tubule, collecting tubule, and collecting duct are essentially impermeable to water unless ADH is present in the body fluids. ADH in the interstitial kidney fluid binds to receptors located in the basal membranes of cells in these portions of the nephron (see figure 14). This coupling of receptor and ADH activates adenyl cyclase, an enzyme associated with the cell membrane, which catalyzes the production of cyclic adenosine monophosphate (cyclic AMP) (Sullivan, 1982). It is the increased cyclic AMP concentration within the cell that increases the permeability of the apical membrane, however, the mechanism by which the permeability changes is not well established. In high concentrations, ADH increases the amount of water absorbed in these nephron segments, thereby causing the formation of a
concentrated urine. Conversely, in the absence of ADH, large amounts of a dilute urine will be formed.

Figure 12: The juxtaglomerular apparatus. (taken from Brown & Stubbs, 1983)

Figure 13: The renin-angiotensin system. (taken from Martini, 1989)
The reabsorption of sodium from the tubular fluid within the distal convoluted tubule, collecting tubule, and collecting duct is controlled by the hormone aldosterone, which is secreted from the adrenal gland when plasma levels of angiotensin II or potassium are elevated. Aldosterone stimulates ion pumps in these portions of the nephron, which then exchange sodium ions for potassium ions. Therefore, aldosterone increases the urinary loss of potassium while reducing this loss of sodium.

Sodium accounts for over 90% of the cations in the extracellular fluid (Guyton, 1984). Due to electroneutrality of the extracellular fluid, the amount of cations automatically controls the number of anions present, so by regulating the concentration of sodium, over 90% of the ions are also controlled. Since the ions account for most of the dissolved species in the bodily fluids, the concentration of sodium is directly related to the fluid osmolality. Therefore, in terms of regulating the body fluid compartments, the renal handling of water and sodium will be the most important factors to consider when modeling kidney function.

In regulating the body fluid compartments, volume is controlled primarily by the arterial pressure, sodium concentration primarily by anti-diuretic hormone, and potassium levels primarily by aldosterone (Guyton & Young in Guyton, Taylor, and Granger, 1975). Increasing the arterial pressure slightly increases renal blood flow and the glomerular filtration rate, however, the urine flow rate can be greatly increased. Furthermore, urine production may cease altogether if the arterial pressure falls below 60 mm Hg. The variation of renal
blood flow, GFR, and urine flow is shown as a function of arterial pressure in figure 15. Therefore, a depleted blood volume, which causes a drop in blood pressure, will decrease urine flow and tend to alleviate the problem. Alternatively, an expanded blood volume, which causes an elevated blood pressure, will increase the urine flow rate thereby decreasing the blood volume.

![Figure 15](image)

**Figure 15**: Effect of acute changes in arterial pressure on the important hemodynamic variables of renal function that relate to renal volume excretion. (taken from Navar & Guyton in Guyton, Taylor, & Granger, 1975)

The secretion rate of ADH and the thirst center response are strongly related to the plasma osmolality. A change in plasma osmolality of 1% doubles the plasma level of ADH and the thirst center is stimulated when the plasma osmolality changes 1 to 2% (Cowley in Guyton, Taylor, and Granger, 1975). An elevated plasma level of ADH is usually believed to increase the blood volume by reducing the urinary water loss and increasing the amount of water ingested. Increasing the plasma ADH level initially will increase in the blood volume but only to a small extent because of the associated increase in arterial pressure and urine flow rate (see figure 15) and the urine which is formed, will have a high solute to water ratio. Consequently, persistently high levels of ADH will cause a slight increase in the blood volume but will also decrease the plasma osmolality due to a high flow rate of concentrated urine. Since sodium is responsible for about 95% of the plasma osmolality, ADH primarily affects the sodium concentration of the extracellular fluid.

Aldosterone causes cells in the later portion of the nephron to absorb sodium from the tubular fluid with the simultaneous secretion of potassium. It would therefore be expected that aldosterone would control the sodium and
potassium levels in the extracellular fluid. However, previous studies have found that aldosterone plays about ten times as much role in the control of potassium concentration as in the control of sodium ion concentration (Guyton & Young in Guyton, Taylor, and Granger, 1975). The reason for this is that the ADH-thirst mechanism is a very potent mechanism for control of sodium ion concentration, so potent that the aldosterone mechanism, in competing with this more potent mechanism, is indeed a very poor competitor (Guyton & Young in Guyton, Taylor, and Granger, 1975).

**Modeling Kidney Function**

Since sodium accounts for over 90% of the cations in the extracellular fluid, and the number of cations is balanced by the number of anions, considering the renal handling sodium and water only should sufficiently describe the relationship between the plasma compartment and kidneys. The following model has been adapted from a previous model of normal renal function in man (Uttamsingh, Leaning, Bushman, Carson, & Finkelstein, 1985).

**Cardiovascular system**

The cardiovascular system consists of a pump, the heart, an assortment of conducting channels, the vessels, and a flowing fluid, the blood. These components are illustrated in figure 16. Arteries carry blood away from the heart and veins return blood to the heart. The blood vessels, which make up the circulatory system, can be sub-divided into parts. Pulmonary vessels bring blood to and from the lungs whereas systemic vessels service the rest of the body.

![Figure 16: The cardiovascular system. (taken from Martini, 1989)](image)
Blood pressure, which affects the glomerular filtration rate, is directly related to the blood volume. An increased blood volume increases the cardiac output and consequently the blood pressure. Additionally, an increased blood volume stretches elastic fibers located in the vessel walls which causes the fibers to contract more powerfully and elevate the blood pressure. A decreased blood volume causes the opposite effects. The experimental relationship between the blood volume, BV, and the mean systemic pressure, MSP, can be described by

\[ MSP = 3.5(BV - 3) \]  \hspace{1cm} (3)

where the mean systemic pressure is the average pressure within the blood vessels from the root of the aorta to the end of the great veins (Uttamsingh, Leaning, Bushman, Carson, & Finkelstein, 1985).

Since angiotensin II is a powerful vasoconstrictor, it will influence the peripheral resistance which is the resistance to blood flow caused by friction with the vessel walls. Short term, neural control of vascular tone is neglected in this model. The relationship between the resistance of the entire circulatory system, or total peripheral resistance (TPR), and the plasma level of angiotensin II (A) can be approximated for humans by the following equations (Uttamsingh, Leaning, Bushman, Carson, & Finkelstein, 1985).

\[ TPR = 19 + 0.037A \quad \text{for} \quad A \leq 27 \text{ng} \]  \hspace{1cm} (4)

\[ TPR = 12.2 + 5.44 \log(A) \quad \text{for} \quad A > 27 \text{ng} \]  \hspace{1cm} (5)

Cardiac output, CO, increases with oxygen consumption which is primarily determined by the metabolic rate. The relationship between cardiac output and oxygen consumption is approximately linear (McArdle, Katch, & Katch, 1986) and is shown in figure 17. This will be described more in detail when increased levels of activity are included in the model. At rest the cardiac output is about 5 liters/min.

Blood flow through vessels can be described by the following relationship.

\[ \text{Blood flow} = \frac{[\text{Pressure (upstream)} - \text{Pressure (downstream)}]}{\text{Resistance}} \]  \hspace{1cm} (6)

The arterial pressure (AP) can be solved for from equation 6.

\[ AP = CO(TPR) \]  \hspace{1cm} (7)
Figure 17: Cardiac output in relation to oxygen consumption during upright exercise in endurance athletes (solid triangles) and sedentary college students prior to (hollow circles) and following (solid circles) 55 days of aerobic training. Arrows represent maximum oxygen consumption for each category. (taken from McArdle, Katch, & Katch, 1986)

Renal function

Glomerular function The forces which control the filtration of plasma fluid into the Bowman's capsule include the capillary hydrostatic pressure and capsular oncotic pressure, which force fluid into the capsular space, and the plasma oncotic pressure and capsular hydrostatic pressure, which force fluid into the arterioles. These latter three pressures remain relatively constant under normal physiological conditions, therefore, the glomerular filtration rate will depend mainly upon the pressure within the arterioles. The following relationship between the arterial pressure and glomerular filtration rate has been previously determined (Goldstein & Rypins, 1992).

\[
GFR = 4.50 - 1.62AP + 0.100(\text{AP})^2 - 1.2 \times 10^{-3}(\text{AP})^3 + 5.73 \times 10^{-6}(\text{AP})^4 - 9.89 \times 10^{-9}(\text{AP})^5
\]  
(8)
Chemical analysis of glomerular filtrate has found that it has approximately the same sodium concentration as plasma (Lote, 1987) so the rate of filtration of sodium into the proximal tubule (FNa) is given by

$$FNa = GFR(PNa)$$  \hspace{1cm} (9)

**Proximal tubule** Electrolytes are actively reabsorbed in this portion of the nephron therefore the rate of sodium reabsorption (SPTR) can be described by a mass transfer coefficient multiplied by the sodium concentration of the tubular fluid.

$$SPTR = GTB(FNa)$$  \hspace{1cm} (10)

where GTB is the glomerular tubular balance or mass transfer coefficient. This coefficient is known to be a function of the sodium concentration of the tubular fluid. Since the concentration of sodium in the proximal tubule is nearly equal to the concentration of sodium in the plasma (PNa), the following linear relationship has been previously derived as a first-order approximation (Uttamsingh, Leaning, Bushman, Carson, & Finkelstein, 1985).

$$GTB = 5.815 - 0.0357PNa$$  \hspace{1cm} (11)

As electrolytes are pumped out of this portion of the tubule, water follows by osmosis. Since sodium makes up the majority of the cations in the filtered fluid, and as sodium is removed from tubule negatively charged ions follow due to an electrical gradient, the fraction of water reabsorbed in the proximal tubule will be nearly equal to the fraction of sodium reabsorbed.

$$EPTR = GTB(GFR)$$  \hspace{1cm} (12)

where EPTR is the rate of water reabsorption in the proximal tubule. From equations 10 and 12, the flow rates of sodium (SFLH) and water (EFLH) into the loop of Henle can be determined.

$$SFLH = FNa - SPTR$$  \hspace{1cm} (13)

$$EFLH = GFR - EPTR$$  \hspace{1cm} (14)

**Loop of Henle** Examinations of the reabsorptive characteristics of sodium and water for the entire loop have demonstrated that the fraction of water reabsorbed (EBLH) is a function of transit time, or an inverse function of flow rate, whereas the fraction of sodium reabsorbed remains fairly constant with flow rate. The following relationships for the rate of reabsorption of sodium (SLHR) and water (ELHR) have been derived for this portion of the nephron (Uttamsingh, Leaning, Bushman, Carson, & Finkelstein, 1985).

$$EBLH = -0.01EFLH + 0.65$$  \hspace{1cm} (15)
The flow rate of water (EFDT) and sodium (SFDT) into the distal tubules is given by

\[
\begin{align*}
EFDT &= EFLH - ELHR \\
SFDT &= SFLH - SLHR
\end{align*}
\]

Distal and collecting tubules In these portions of the nephron, the amount of water reabsorbed is controlled by antidiuretic hormone (ADH) and the amount of sodium reabsorbed by aldosterone (ALD). Using data from previous experiments, relationships for the rate of water reabsorbed (EDTR) and the rate of sodium reabsorbed (SDTR) have been derived (Goldstein & Rypins, 1992).

\[
\begin{align*}
EDTR &= EFDT\left[0.0417 - 0.400ADH + 0.637(ADH)^2 - 0.222(ADH)^3 + 0.0345(ADH)^4 - 0.00254(ADH)^5 + 7.25 \times 10^{-5}(ADH)^6\right] \\
SDTR &= SFDT \left[0.572 + 0.00195(ALD) + 5.15 \times 10^{-5}(ALD)^2 - 7.98 \times 10^{-7}(ALD)^3 + 4.93 \times 10^{-9}(ALD)^4 - 1.69 \times 10^{-11}(ALD)^5 + 3.52 \times 10^{-14}(ALD)^6 - 4.56 \times 10^{-17}(ALD)^7 + 3.61 \times 10^{-20}(ALD)^8 - 1.59 \times 10^{-23}(ALD)^9 + 3.02 \times 10^{-27}(ALD)^{10}\right]
\end{align*}
\]

Urine flow (UFL) is then given by

\[
UFL = EFDT - EDTR
\]

and the urinary excretion rate of sodium (UNa) is given by

\[
UNa = SFDT - SDTR
\]

Note that these equation predict that ADH can produce large percentage changes in fluid reabsorption whereas aldosterone has only a small modulating effect on sodium reabsorption. This agrees with the discussion at the end of the previous section (Description of the kidney) which stated that ADH is the primary controller of sodium in the extracellular fluid and the urinary sodium excretion rate. However, a full analysis requires the inclusion of aldosterone.

Hormonal systems

Having derived equations for the reabsorption of sodium and water in the latter portions of the nephron, which depend upon ADH and aldosterone, the levels of these hormones must now be estimated.
Control of ADH concentration  Plasma ADH concentration is determined by three factors: the rate of ADH release from the posterior pituitary gland which depends upon signals from osmoreceptors and stretch receptors, the rate of clearance of ADH from the body by the liver and kidneys, and the volume in which the ADH is dispersed. As seen in figure 6, plasma ADH levels, and consequently the ADH release rate, increase as plasma osmolality increases and blood volume decreases. Equations for ADH release as a function of plasma osmolality (ADHSP) (DeHaven & Shapiro, 1970) and extracellular compartment volume (ADHSV) (Uttamsingh, Leaning, Bushman, Carson, & Finkelstein, 1985) have been derived from experimental data.

\[
ADHSP = 0.833PNa - 117.45 \text{ for } PNa \geq 141.9 \text{ mosm/l (24)}
\]

\[
ADHSP = 0.06PNa - 7.83 \text{ for } PNa < 141.9 \text{ mosm/l (25)}
\]

\[
ADHSV = 0.0 \text{ for } DWV > 1.8 \text{ (26)}
\]

\[
ADHSV = 0.15 - 0.083DWV \text{ for } 1.8 > DWV \geq 1.0 \text{ (27)}
\]

\[
ADHSV = 0.813 - 0.75DWV \text{ for } 1.0 > DWV \geq -1.2 \text{ (28)}
\]

\[
ADHSV = 1.71 \text{ for } -1.2 > DWV \text{ (29)}
\]

where DWV is the deviation of the extracellular compartment volume (E) from the normal value (EN).

\[
DWV = E - EN \text{ (30)}
\]

The signals for ADH release in response to variations in plasma osmolality and blood volume are additive if both signals tend to increase the ADH release rate (i.e. increased plasma osmolality with decreased blood volume). However, if both the plasma osmolality and blood volume are above normal, the signal for blood volume will be the primary signal. Recall that volume overrides tonicity. For this case the net rate of ADH (ADHS) release is given by the following equations (Uttamsingh, Leaning, Bushman, Carson, & Finkelstein, 1985).

\[
ADHS = \frac{17.0(DWV)(ADHSV) + ADHSP}{17.0(DWV) + 1.0} \text{ for } POS > 299.6 \text{ mosm/l and } DWV > 2.0 \text{ (31)}
\]

\[
ADHS = \frac{[33.0(DWV) - 32.0]ADHSV + ADHSP}{33.0(DWV) - 31.0} \text{ for } POS > 299.6 \text{ mosm/l and } 1.0 \leq DWV \leq 2.0 \text{ (32)}
\]

For all other cases
ADHS = \frac{ADHSV + ADHSP}{2.0} \quad (33)

The rate of clearance of ADH from the plasma (DADH) has been found to be related to the plasma concentration of ADH (Uttamsingh, Leaning, Bushman, Carson, & Finkelstein, 1985).

\begin{align*}
\text{DADH} &= 0.206 \quad \text{for ADH} > 4.0 \text{ munits/l} \quad (34) \\
\text{DADH} &= 0.374 - 0.042 \text{ ADH} \quad \text{for ADH} \leq 4.0 \text{ munits/l} \quad (35)
\end{align*}

where the clearance rate of a substance is defined as the amount of blood completely cleared of the substance per unit time. For example, a clearance rate for ADH of 0.206 l/min means that ADH is completely removed from 0.206 liters of blood every minute.

Previous studies have found that ADH is confined mainly to the plasma compartment, therefore, the volume that ADH is distributed into is equal to the plasma volume (PV) (Uttamsingh, Leaning, Bushman, Carson, & Finkelstein, 1985). A material balance on ADH yields

\[
(PV) \frac{d(ADH)}{dt} = ADHS - DADH(ADH) \quad (36)
\]

Control of aldosterone concentration. Aldosterone release is one of the final consequences of the renin/angiotensin system, the function of which is to provide feedback control on the rates of sodium and potassium excretion, and thereby influence the volume of the extracellular and intracellular compartments (see figure 13). In absence of more explicit data, a linear relationship has been postulated for the rate of renin release (RS) as a function of the amount of sodium entering the distal tubule (Uttamsingh, Leaning, Bushman, Carson, & Finkelstein, 1985).

\[
RS = 0.0163 - 0.0093SFDT \quad (37)
\]

This equation is consistent with the fact that the macula densa, which secretes renin into the plasma compartment, monitors fluid within the distal tubule. Renin is removed from the circulation on passage through the liver with a clearance rate of approximately 0.135 l/min (Uttamsingh, Leaning, Bushman, Carson, & Finkelstein, 1985). A material balance on renin yields

\[
(PV) \frac{dR}{dt} = RS - 0.135R \quad (38)
\]

where R is the plasma concentration of renin.

Renin catalyses the reaction which converts circulating angiotensinogen to angiotensin I. Angiotensin I is rapidly converted to angiotensin II by enzymes
in the lungs. The following equation has been derived for the rate of formation of angiotensin II (AS) (Uttamsingh, Leaning, Bushman, Carson, & Finkelstein, 1985).

\[ AS = 583.3R(PV) \]  

(39)

The rate of clearance of angiotensin II from the plasma has been found to be approximately 4.04 l/min (Uttamsingh, Leaning, Bushman, Carson, & Finkelstein, 1985). A material balance on angiotensin II yields

\[ (PV) \frac{dA}{dt} = AS - 4.04A \]  

(40)

The major factor which regulates the release of aldosterone from the adrenal gland is the plasma concentration of angiotensin II. From previous animal studies, the following relationships for the rate of aldosterone release (ALS) as a function of plasma angiotensin II concentration have been derived (Uttamsingh, Leaning, Bushman, Carson, & Finkelstein, 1985).

\[ ALS = 0.75A + 7.76 \quad \text{for } A < 18 \text{ ng/l} \]  

(41)

\[ ALS = 3.32A - 38.5 \quad \text{for } 18 \leq A < 34.0 \]  

(42)

\[ ALS = 0.585A + 54.6 \quad \text{for } A \geq 34.0 \]  

(43)

The clearance rate of aldosterone from the plasma has been found to be approximately 0.62 l/min (Uttamsingh, Leaning, Bushman, Carson, & Finkelstein, 1985) which leads to the following material balance.

\[ (PV) \frac{dALD}{dt} = ALS - 0.62ALD \]  

(44)

Sodium and potassium balance Since the overall model of the body water compartments is concerned mainly with water and sodium, recall that sodium ions account for almost 95% of the cations in the extracellular fluid, the present model will assume that the extracellular potassium concentration remains constant at a typical value of 4 meq/l. The timed average daily ingestion rate of sodium (SODMIN) and the urinary excretion rate of sodium (UNa) will determine the amount of sodium in the extracellular fluid (TENa).

\[ \frac{d(TENA)}{dt} = SODMIN - UNa \]  

(45)

Since cellular membranes are relatively impermeable to electrolytes when compared to water, the amount of sodium in the intracellular fluid will remain constant.
\[ \frac{d(TINA)}{dt} = 0 \quad (46) \]

The concentration of sodium in the extracellular compartment (PNa) and the intracellular compartment (INa) will then be given by

\[ PNa = \frac{TENA}{E} \quad (47) \]

\[ INa = \frac{TINA}{I} \quad (48) \]

where \( E \) is the extracellular fluid volume and \( I \) is the intracellular fluid volume.

**Total body water balance** A mass balance on total body water (W) gives

\[ \frac{dW}{dt} = FLUMIN - UFL \quad (48) \]

where \( FLUMIN \) is the flow rate of water into the body. Assuming instantaneous osmotic equilibration between the intracellular (I) and extracellular (E) compartments

\[ \frac{(TENA + TEC)}{E} = \frac{(TINA + TIC)}{I} \quad (50) \]

where \( TEC \) and \( TIC \) are constants which represent the other dissolved species within the extracellular and intracellular compartments, respectively.

Instantaneous osmotic equilibrium between the extracellular and intracellular compartments will be assumed only when testing the kidney model. Since the mass transfer coefficient for the transfer of water across cellular membranes is known (see figure 1), the overall model of the body water compartments will not assume instantaneous equilibrium.

Since \( W = E + I \), the size of the intracellular and extracellular compartments can be derived from equation (50).

\[ E = \frac{W}{1 + \frac{(TINA + TIC)}{(TENA + TEC)}} \quad (51) \]

\[ I = \frac{W}{1 + \frac{(TENA + TEC)}{(TINA + TIC)}} \quad (52) \]

**Testing the kidney model**

To test the validity of the proposed kidney model, results predicted by the model will be compared to actual data involving injected or ingested fluids and subsequent urine flow rates. Under these conditions, water enters the body through the plasma compartment and leaves the body through the formation of urine. Intravenously injected fluids enter the plasma compartment immediately whereas ingested fluid must first be absorbed by the intestines. The rate constant for water absorption from the gastrointestinal tract has been estimated
to be 3.33 hr\(^{-1}\) (see figure 1). Amounts of water produced by cellular metabolism and excreted through sensible and insensible perspiration will be relatively small and consequently neglected for this case. Comparison of the model simulation to actual data following the ingestion of 1 liter of water is shown in figure 18. In figure 19, the model simulation is shown with actual data following the intravenous infusion of hypertonic saline.

**Figure 18:** Results from the model simulation (solid line) and experimental data (solid circles) following ingestion of 1 liter of water. (actual data taken from Baldes & Smirk, 1934)

**Figure 19:** Results from the model simulation (solid line) and experimental data (solid circles) following infusion of 9.8 g/min of a 10% NaCl solution for 65 minutes. (actual data taken from Dean & McCance, 1949)
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</tr>
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<td>CO</td>
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<td>E_N</td>
<td>normal extracellular fluid volume (l)</td>
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<td>Hct</td>
<td>hematocrit (unitless)</td>
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<td>I</td>
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<td>Variable</td>
<td>Description</td>
<td>Value</td>
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<td>----------</td>
<td>-------------</td>
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<td>total extracellular sodium (mEq)</td>
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<td>TIC</td>
<td>total intracellular osmotic components other than sodium</td>
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<td>TPR</td>
<td>total peripheral resistance (torr/l-min)</td>
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<td>UFL</td>
<td>urine flow rate (ml/min)</td>
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<td>UNa</td>
<td>rate of excretion of sodium (mEq/min)</td>
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<td>VR</td>
<td>venous return (l/min)</td>
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<td>W</td>
<td>total body water (l)</td>
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<td>( \pi_c )</td>
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<tr>
<td>( \pi_i )</td>
<td>interstitial oncotic pressure (torr)</td>
<td>0.0</td>
</tr>
</tbody>
</table>

(* subject dependent variable)
APPENDIX 3
Modifications to an interactive model of the human body during exercise: With special emphasis on thermoregulation

by

Megan Kay Scherb

A Creative Component Submitted to the Graduate Faculty in Partial Fulfillment of the Requirements for the Degree of MASTER OF ENGINEERING

Department: Chemical Engineering
Major: Chemical Engineering

Iowa State University
Ames, Iowa

1992
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<tr>
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</tr>
<tr>
<td>VO2&lt;sub&gt;max&lt;/sub&gt;</td>
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INTRODUCTION

Since 1988 an interactive computer model of the human body during exercise has been under development by a number of undergraduate students in the Department of Chemical Engineering at Iowa State University. The program, written under the direction of Dr. Richard C. Seagrave, uses physical characteristics of the user, environmental conditions and activity information to predict the onset of hypothermia, hyperthermia, dehydration, or exhaustion for various levels and durations of a specified exercise. The program, however, was severely limited in predicting the onset of dehydration due to the lack of sophistication with which the program predicted sweat rate and its relationship to sensible water loss, degree of acclimatization, and level of physical training. Additionally, it was not known whether sweat rate also depended on age and gender. For these reasons, the goal of this creative component was to modify the program in the above mentioned areas by applying known information and empirical relationships from literature. Furthermore, a secondary goal was to improve the consistency with which the program was written by modifying user input statements and improving the efficiency and logic of the program calculations.

Principle Mechanisms of Heat Transfer During Exercise

The human body exchanges heat with the environment by four basic mechanisms; radiation, conduction, convection, and evaporation. These mechanisms are the same whether heat is imposed to the body metabolically, as in exercise, or environmentally. Radiative heat transfer occurs by the
transmission of electromagnetic heat waves between objects without the involvement of molecular contact. Heat transfer by radiation may result in either heat loss or gain depending on conditions in the environment. For example, when a person is cooler than the surroundings, he/she will absorb radiant heat energy. Conversely, when a person is warmer than the surroundings, he/she will lose heat to the environment.

Conductive heat transfer occurs when heat energy is transferred between objects by direct physical contact. Approximately three percent of a person's total heat loss at rest in a room temperature environment occurs via this mechanism (Brooks et al., 1987). Therefore, during exercise, heat loss by means of conduction is considered negligible. Conduction also occurs within the body itself as heat is conducted from the inner core to the skin or vice versa depending on the temperature gradient.

Convective heat transfer, a form of conduction of heat to a fluid, occurs via the circulation of air molecules adjacent to the skin. As air molecules are warmed, they become less dense, rise away from the body, and are displaced by cooler molecules. During exercise in air, heat loss due to convection is influenced by both the velocity of the athlete and the wind.

Finally, evaporative heat transfer occurs by way of the change of liquid water on the surface of the skin to gaseous water vapor in the environment. Evaporative heat transfer is the most important mechanism of heat transfer during exercise. In evaporation, heat is transferred from the body to water on the surface of the skin. When the water has gained sufficient energy, approximately 0.58 kilocalories per gram of water depending on the skin temperature, it vaporizes and heat is removed from the skin. It is important to note that unlike
radiation, conduction, and convection which occur because of temperature gradients, evaporation is driven by a gradient in the vapor pressure. During exercise, this fact is especially salient in an environment where the ambient temperature and relative humidity are high. In this situation, the body gains heat by convection and radiation, and evaporation becomes the only mechanism by which the body can lose heat. However, when the relative humidity is high, temperature regulation becomes more difficult because the vapor pressure of the air is close to that of moist skin. Therefore, the rate of evaporation is greatly reduced. Sweating in these environments results in sensible water loss that can lead to dehydration without a cooling effect to the skin (McArdle, 1986).

In addition to the four basic mechanisms of heat transfer, heat loss via respiration (a combined mechanism) must also be considered during exercise. Respiratory energy losses occur via the evaporation of water in the lungs and respiratory tract as well as by the convective cooling effect of breathing.

**Physiological Control of Heat Transfer During Exercise**

There are two principle mechanisms by which the body can control the balance between heat production and heat loss during exercise. First, the body can change its surface temperature by altering blood flow to the skin. When blood vessels to the skin dilate (vasodilation), warm blood from the core of the body is brought to the surface where heat is lost via radiation and convection. Conversely, when these blood vessels constrict (vasoconstriction), heat is conserved in the inner core and less heat is lost to the surroundings. Second, the body can regulate the rate of sweating.
These physiological mechanisms are governed by the thermoregulatory center located in the hypothalamus. The hypothalamus functions as the body's thermostat by maintaining the body core temperature within a narrow range around its set-point of 37 degrees Celsius. When the core temperature increases above or decreases below this set-point, the hypothalamus initiates a response to increase or decrease heat production or facilitate heat loss.

The anterior hypothalamus reacts to increases in body heat, while the posterior hypothalamus is responsible for regulating reactions to a cold environment (Åstrand, 1986). Therefore, when the anterior hypothalamus senses an increase in core temperature above the set-point, it stimulates the sweat glands resulting in increased evaporative cooling. In addition, normal vasoconstriction is inhibited and blood flow to the skin is promoted. Conversely, when the posterior hypothalamus senses a decrease in core temperature below the set-point, the vasomotor center causes the peripheral blood vessels to constrict and, if the core temperature drops significantly, shivering commences.

**LITERATURE REVIEW**

Numerous reviews on the subject of exercise thermoregulation have been written in the past two and a half decades. For additional information on this subject, the reader is referred to two of the more recent reviews, Gisolfi et al. (1984) and Buskirk (1977).
Before specifically addressing the areas of interest of this work, it is necessary to provide background information regarding the relationships of physiological variables such as skin and core temperature to effector responses.

Since body tissue temperatures differ based on local rates of heat production and heat exchange as well as heat transfer between locations in the body, definitions are necessary to specify these variations. This paper will only consider the differences between core and skin temperatures.

Deep-body, or core temperature, is usually measured at one of three locations; rectum, esophagus or the tympanic membrane. These estimates of core temperature are not wholly equivalent to each other, but have been shown to vary in parallel and are thus effective measures of relative changes (Åstrand, 1986).

Skin temperature can be determined by assigning weights to specific skin temperature measurements at various locations on the body in proportion to the fraction of the body's total surface area represented by that location. The most commonly used formula which was developed by Hardy and Dubois follows (Åstrand, 1986):

\[
Ts = 0.07 \, T_{\text{head}} + 0.14 \, T_{\text{arm}} + 0.05 \, T_{\text{hand}} + 0.07 \, T_{\text{ft}} + 0.13 \, T_{\text{leg}} + 0.19 \, T_{\text{th}} + 0.35 \, T_{\text{tr}}
\]

It is generally accepted that skin temperature is dependent on the environmental temperature surrounding the skin and is relatively independent of exercise load. Conversely, the core temperature is independent of the ambient temperature and is largely dependent on the relative exercise load, as expressed as a percentage of maximum oxygen uptake, \(VO_2\text{max}\) (Åstrand, 1986). For example, if two people perform the same absolute work load but have different
maximum oxygen uptakes, the person with the lower VO_{2\text{max}} will experience a
greater rise in core temperature. However, if the same two people exercise at the
same percentage VO_{2\text{max}}, their core temperatures should increase by
approximately the same amount.

In contrast, sweat rate is more closely related to the absolute work load
than to relative work load (Buskirk, 1977). Furthermore, body core temperature
can be as much as ten times more influential on sweat rate than skin
temperature (Wyndham, 1965). Wyss et al. (1974) supported this finding by
concluding that sweat rate was virtually independent of steady state skin
temperature and that body core temperature was the dominant factor in skin
blood flow, heart rate and sweat rate determination. Additionally, Davis et al.
(1976) concluded that skin temperature was independent of evaporative sweat
loss and relative exercise load.

Effects of Heat Acclimatization and Physical Training

Adaptations to sweating are typically caused by both an exercise and a heat
effect (Nadel et al., 1974). However, it has been shown that these adaptations
occur via two different mechanisms. Physical training results in enhanced
sweating at a given level of central drive. The increased metabolic rate during
training increases thermoregulatory demand and induces an increased
peripheral sensitivity of the sweat glands (Nadel et al., 1974). In other words,
physical training increases the slope of the sweat rate versus core temperature
relationship. Nadel et al. (1974) noted an increase of 67 percent in this slope as a
result of ten consecutive days of one hour exercise at a relative exercise intensity
of 70 to 80 percent VO_{2\text{max}}.
In contrast to physical training, heat acclimatization lowers the threshold core temperature at which sweating starts, increases sweating capacity, decreases skin temperature, and reduces the heart rate (Nadel et al., 1974; Fortney and Senay, 1979; Frye and Kamon, 1981; Horstman and Christensen, 1982). Nadel et al. (1974) noted a 0.3 degree Celsius decrease in the threshold core temperature, while Brooks et al. (1987) noted a three fold increase in sweating capacity from 1.5 to 4 kilograms per hour as a result of heat acclimatization.

According to Bass (1963), the acclimatization process begins on the first day of exposure and is well developed in 4 - 7 days. It can be induced by short intermittent periods of work or exercise in the heat, and is retained during periods of no exposure for about two weeks. Additionally, persons in good physical condition tend to acclimatize more rapidly than unfit subjects. However, physical training cannot replace heat acclimatization (Strydom et al., 1966; Gisolfi and Cohen, 1979). Training results in partial acclimatization that may improve heat tolerance by up to 50 percent (Gisolfi and Cohen, 1979), but training alone cannot substitute for exercise in the heat.

Figure 1 of Appendix A illustrates schematically the effects on sweat rate of both heat acclimatization and physical training discussed above.

**Effect of Dehydration**

As noted previously, athletes who exercise for prolonged periods can lose up to four liters of body fluids in the form of sweat and can experience a total weight loss of seven to eight percent of body weight during an endurance event such as a marathon (Lamb, 1984). Since the body contains only 40 liters of fluid, of which 5 liters are blood, loss of a large portion of this fluid results in a decrease
in blood volume, cardiac output, and blood pressure. A loss of two to three liters of body fluids during exercise causes a reduction in sweating which results in an increase in core temperature (Nadel, 1979). The data of Greenleaf and Castle (1971) show that at a level of dehydration equivalent to loss of fluid equal to about five percent body weight, core temperature increases significantly due to inadequate sweating. Nadel (1979) further postulates that dehydration causes a reduction in the sensitivity of the sweat rate. In other words, dehydration decreases the slope of the sweat rate versus core temperature relationship. Please refer to Figure 2 in Appendix A for a schematic illustration of this effect.

**Effects of Age and Gender**

Although the experimental data are limited, the evidence in literature suggests that there are few differences in the thermoregulatory responses which can be ascribed to age and gender (Davies, 1979). Studies by Drinkwater et al. (1982) show that the functional capacity of the sweating mechanism in healthy older women does not decrease with age. Additionally, no differences were reported between the sexes with respect to sweat rate or efficiency in dry heat, but women maintained a higher sweating efficiency in humid heat (Frye and Kamon, 1983). This increased efficiency allows women to conserve body water while maintaining a similar core temperature to that of men. In other words, for the same degree of evaporative cooling, women secrete less wasteful sweat. When men and women of similar fitness levels are compared, the previously reported gender differences with respect to heat exposure disappear (Avellini et al., 1980). Furthermore, Horstman and Christensen (1982) concluded that active
men and women compared at the same level of relative exercise intensity performed exercise equally well in dry heat.

PROGRAM MODIFICATIONS

As many individuals had modified various parts of the program over the past four years, the program contained inconsistencies. Therefore, to improve the program two types of modifications were made; organizational changes and functional changes. Organizational changes involve modifications that changed the order in which the program performed specific operations. Functional changes are modifications that improved specific functions of the program, i.e. the consistency of wording in the user input statements, or changes to specific calculations.

In addition to the organizational and functional changes, specific modifications were made to the program as a result of information found in the literature. Modifications were made to account for the following effects; physical training, acclimatization to heat, and dehydration. Additionally, a modification was made to correct the program for its premature prediction of the onset of dehydration. Please refer to the program listing in Appendix C for an updated version of the program.

Organizational Changes

Modifications which improve the flow and efficiency of the program are described below. Please refer to the flow diagram in Figure 3 of Appendix A for more detailed information regarding the general organization of the program.
(1) The main program was divided into four sections: (a) user input, (b) calculation of constants and initial values, (c) calculation of material and energy balances, (d) calls for subroutines to calculate fuel usage, print summary information, and plot data.

(2) The section which prints summary information was rewritten as a subroutine (PRINTAB).

(3) The order of the user input statement was improved. The program now asks the user to input information in the following order; personal physical characteristics, environmental conditions and activity information.

(4) All format statements were moved to the end of the program or subroutine to which they apply.

Functional Changes

In order to improve the consistency of the program and the manner in which certain calculations are performed, the following modifications were made:

(1) The number of variables and constants defined within the program was increased.

(2) The language of the variable and constant definitions was improved to maintain consistent use of language throughout the list. Units of the variable or constant were also included in the definition.

(3) Additional tests were added in the user input section to ensure valid information is entered.

(4) Screen stops were added for all output printed to the screen.

(5) The consistency of the language in the user input section was improved.

(6) All FORTRAN ‘print’ statements were changed to ‘write’ statements to facilitate future modifications to input/output formats.
(7) A question regarding the user's gender was added to the input section.

(8) A subroutine PLOTDATA was added to plot program results versus time. These results include: core temperature, heat loss, body weight, and skin temperature.

(9) Revised expressions were added to calculate basal metabolic rate (Olson, 1992). The expressions follow:

Female: \[ \text{BASAL (kcal/day)} = 655.1 + 9.563 \text{ MKG} + 1.850 \text{ HT} - 4.676 \text{ AGE} \]

Male: \[ \text{BASAL (kcal/day)} = 66.5 + 13.75 \text{ MKG} + 5.003 \text{ HT} - 6.775 \text{ AGE} \]

Correction for Premature Prediction of Dehydration

Several simulations of the program at various levels and durations of exercise indicated that the program prematurely predicted the onset of dehydration. This occurred because the sweat rate equations estimate a high amount of sensible water loss as run-off. It is postulated by Seagrave that high quantities of run-off are estimated because Nielsen's (1969) sweat rate measurements were taken after steady state temperatures had been achieved. However, the program uses these sweat rate estimations in an unsteady state capacity. According to Kerslake (1963), loss of sweat by dripping does not begin until the sweat rate has reached one-third of the maximum evaporative capacity. Therefore, to improve the program, it is necessary to determine the point at which sweat run-off commences.

The program was modified by adding a statement to calculate one-third maximum evaporative capacity, the point at which sweating begins. If the sweat rate has not reached this point, the program uses the rate of evaporative mass transfer as an estimation of the water loss. However, if the sweat rate has
reached one-third maximum evaporative capacity, the program uses the water loss calculated by the WATERLOSS subroutine.

Effect of Heat Acclimatization

Before any modifications could be made to the program, the existing sweat rate equations in the WATERLOSS subroutine had to be modified. These equations had been extrapolated by Woodard from data given in Nielson (1969). These relationships expressed sweat rate as a function of skin temperature at given metabolic rates and skin temperatures. These relationships were modified to express sweat rate as function of body core temperature, rather than skin temperature. To make this modification, core temperature, skin temperature and sweat rate data were generated using the program. Subsequently, linear regressions were performed on the data to generate equations which expressed sweat rate as a function of body core temperature for given skin temperatures and metabolic rates. Please see Table 1 in Appendix B for the revised sweat rate equations.

To account for heat acclimatization, two modifications were made. First, if the athlete is both acclimatized to heat and physically trained, the maximum sweat rate (MAXSR) is set to 4.0 kilograms per hour and the set-point of the body core temperature is lowered by 0.3 degrees Celsius to 36.7 degrees. However, if the athlete is not acclimated to heat, the maximum sweat rate is set to 1.5 kilograms per hour and the body core temperature is set to the standard set-point of 37 degrees Celsius.
Effect of Physical Training

To modify the program for the effect of physical training on sweat rate, statements were added to section 2 of the main program to change the slope of the sweat rate equations that appear in subroutine WATERLOSS. If an athlete is trained, GAIN1 is set to 1.65, or the slope of the sweat rate equation is increased by 65%. However, if the athlete is not trained, there is no change to the slope of the sweat rate equation and GAIN1 is set equal to 1.0.

Effect of Dehydration

To account for the effect of dehydration, statements were added to adjust the slope of the sweat rate equations in the WATERLOSS subroutine as dehydration progresses. The gain constants were chosen somewhat arbitrarily due to the lack of specific numerical information provided in the literature. However, statements were added in section 3 of the main program to adjust the slope of the sweat rate equations in the following manner: when the body is less than one percent dehydrated, no change is made to the slope of the sweat rate equation or GAIN2 is set to 1.0. When total dehydration is between one and two percent, the slope of the sweat rate equation is decreased by five percent or GAIN2 is set to 0.95. For increasing percentages of dehydration, the slope is decreased in ten percent increments to a maximum level of dehydration of five percent of the total body weight. At this level of dehydration, the slope of the sweat rate equation has been decreased a total of 35 percent.
RECOMMENDATIONS FOR FUTURE WORK

In order to improve the program the following recommendations are provided as areas of future work:

(1) Modify the program to account for partial acclimatization to heat.

(2) Modify the program to account for the fact that women are less wasteful sweaters.

(3) Add sweat rate equations for metabolic rates greater than 1700 kilocalories per hour.

(4) Further quantify the effect of dehydration on sweat rate.

(5) Determine the effect of cold acclimatization on exercise thermoregulation.

(6) Quantify the effect of transverse wind velocity on the total velocity.

(7) Quantify maximum glucose usage for all exercises included in the program.
LITERATURE CITED


Olson, C. E., 1992. Integration of human material and energy balance in life support system descriptions, AeroE 499 Project, Iowa State University, Ames, IA.

Woodard, L. K., 1988. Model of the human body during exercise, ChE 490 Project, Iowa State University, Ames, IA.


ACKNOWLEDGMENTS

I would like to thank Dr. Richard C. Seagrave for his assistance and thought provoking advice during the period of this work. I would also like to thank Dr. Dean L. Ulrichson and Dr. Paula C. Morrow for serving on my committee.

Additionally, I would like to thank the NASA Advanced Life Support Division and Honeywell Foundation for the financial support provided throughout the duration of my graduate study at Iowa State University.
APPENDIX A:
Figure 1: Chest sweating rate as a function of esophageal temperature prior to acclimatization, following 10 days of physical training and following 10 day of heat acclimatization. (Taken from Nadel, 1974)

Figure 2: A schematic representation of one possible effect of dehydration on the sweating rate as a function of internal temperature. (Taken from Nadel, 1979)
Figure 3: Flow Diagram of a Model of the Human Body During Exercise
<table>
<thead>
<tr>
<th>Metabolic Rate (kcal/hour)</th>
<th>TSC Constraint(s) (Deg. C)</th>
<th>Regression Equation:</th>
</tr>
</thead>
<tbody>
<tr>
<td>Less than 300</td>
<td>None</td>
<td>DSWEAT = 0.046122 TBC - 1.16399</td>
</tr>
<tr>
<td>301 - 500</td>
<td>&lt;= 33.0</td>
<td>DSWEAT = 0.063091 TBC - 1.53415</td>
</tr>
<tr>
<td></td>
<td>&gt; 33.0</td>
<td>DSWEAT = 0.295611 TBC - 9.53531</td>
</tr>
<tr>
<td>501 - 700</td>
<td>&lt;= 33.4</td>
<td>DSWEAT = 0.091916 TBC - 2.21063</td>
</tr>
<tr>
<td></td>
<td>&gt; 33.4</td>
<td>DSWEAT = 0.512109 TBC - 16.9846</td>
</tr>
<tr>
<td>701 - 900</td>
<td>&lt;= 32.0</td>
<td>DSWEAT = 0.223172 TBC - 6.03852</td>
</tr>
<tr>
<td></td>
<td>&gt; 32.0</td>
<td>DSWEAT = 0.390976 TBC - 11.8576</td>
</tr>
<tr>
<td>901 - 1100</td>
<td>None</td>
<td>DSWEAT = 0.256577 TBC - 5.75055</td>
</tr>
<tr>
<td>1101 - 1300</td>
<td>&lt;= 29.0</td>
<td>DSWEAT = 0.439115 TBC - 12.3871</td>
</tr>
<tr>
<td></td>
<td>&gt; 29.0</td>
<td>DSWEAT = 0.314289 TBC - 8.09200</td>
</tr>
<tr>
<td>1301 - 1500</td>
<td>&lt;= 28.6</td>
<td>DSWEAT = 0.699080 TBC - 21.2381</td>
</tr>
<tr>
<td></td>
<td>&gt; 28.6</td>
<td>DSWEAT = 0.227225 TBC - 4.99327</td>
</tr>
<tr>
<td>1501 - 1700</td>
<td>None</td>
<td>DSWEAT = 0.053090 TBC + 1.639657</td>
</tr>
</tbody>
</table>
MODEL OF THE HUMAN BODY AND EXERCISE

REAL Y,N,WTLBS, MKG, TIC, WMPH, WMETPS, BASAL, GAIN1, GAIN2
REAL RHUM, MDOT, VMPH, VMETPS, VELTOT, TBC, HV, PA, YA, AIRMOL
REAL VAIrl, VOMAX, MILE, IPVO, PVO, SGLU, TSC, DSWEAT, FS, SWEAT, DTB
REAL TSWEAT, DGLU, GLU, QRES, QEVAP, QCONV, QRAD, QTOT, CB, AC, HC, CA
REAL KG, AV, AR, HR, TIF, VOL, LRA, CLO, K, YARRAY(40), WATERLOSS
REAL CHFRAC, LFRAC, CHEQ, LIPEQ, TOTCAL, CHCAL, LIPCAL, CALEQ
REAL LIPIUSED, CHUSED, BCM, BCM, QUAD, HAM, BICEP, TRICEP, CALF
REAL HTVAP, EMAX, DRIP, MAXSR, TS
CHARACTER ANS*4, C*1, WW*3, GENDER*1, ICASE*1
CHARACTER NAME*50, EXDESC*50, CLOTHES*50, BOXANS*15, FRAME*15
INTEGER TT, NUMDAT, ICLO, CHOICE, I, XARRAY(40)
INTEGER AGE, FT, IN, MIN, SEC, TTMAX, ACTLEV, ACCLEV
COMMON NAME, FT, IN, WTLBS, AGE, CLOTHES, MIN, SEC, ACTLEV, EXDESC
COMMON MDOT, VOL, VMPH, TIF, RHUM, WMPH, TS, CBW, CBT, FTSC, TOTCAL
COMMON CHCAL, LIPCAL, CHUSED, LIPIUSED, TIME

write(*,2) 'This program is designed to inform well trained'
write(*,*) 'and novice athletes how long they may exercise at'
write(*,*) 'a certain level before one of the following'
write(*,*) 'conditons occur:'
write(*,1)
write(*,3) 'a) hyperthermia: estimated core temperature'
write(*,3) 'surpasses 40 Deg. C (104 Deg. F)'
write(*,3) 'b) dehydration: loss of body water exceeds 5%'
write(*,3) 'of body weight'
write(*,3) 'c) estimated muscle glycogen supply exceeded'
write(*,3) 'd) hypothermia: estimated core temperature'
write(*,3) 'drops below 34 Deg. C (92 Deg. F)'
write(*,1)

PROGRAM VARIABLES/CONSTANTS

<table>
<thead>
<tr>
<th>VARIABLE/CONSTANT</th>
<th>DESCRIPTION</th>
</tr>
</thead>
<tbody>
<tr>
<td>AC</td>
<td>Body area available for convection (94% total body area, m^2)</td>
</tr>
<tr>
<td>ACCLEV</td>
<td>Level of acclimatization of user (heat, cold, unacclimatized)</td>
</tr>
<tr>
<td>ACTLEV</td>
<td>Activity level of user (trained/untrained)</td>
</tr>
<tr>
<td>AGE</td>
<td>Age of the athlete (years)</td>
</tr>
<tr>
<td>Abbreviation</td>
<td>Description</td>
</tr>
<tr>
<td>--------------</td>
<td>-------------</td>
</tr>
<tr>
<td>AIRMOL</td>
<td>Air intake conversion (mol/hr)</td>
</tr>
<tr>
<td>ANS</td>
<td>Answer to question (yes/no)</td>
</tr>
<tr>
<td>AR</td>
<td>Body area available for radiation (78% total body area, m^2)</td>
</tr>
<tr>
<td>AV</td>
<td>Body area available for evaporation (15% total body area, m^2)</td>
</tr>
<tr>
<td>BASAL</td>
<td>Basal metabolic rate (kcal/min)</td>
</tr>
<tr>
<td>BCM</td>
<td>Volume of body (m^3)</td>
</tr>
<tr>
<td>BCMB</td>
<td>Volume of body taking into account body build (m^3)</td>
</tr>
<tr>
<td>HAM</td>
<td>Glycogen content in hamstrings (mmol)</td>
</tr>
<tr>
<td>BICEP</td>
<td>Glycogen content in bicep (mmol)</td>
</tr>
<tr>
<td>BSA</td>
<td>Surface area of body as a function of athlete height and mass (m^2)</td>
</tr>
<tr>
<td>CA</td>
<td>Heat capacity of air (kcal/gmole*Deg. C)</td>
</tr>
<tr>
<td>CALF</td>
<td>Glycogen content in calf (mmol)</td>
</tr>
<tr>
<td>CB</td>
<td>Heat capacity of the body (kcal/kg*Deg. C)</td>
</tr>
<tr>
<td>CBT</td>
<td>Current body temperature (Deg. F)</td>
</tr>
<tr>
<td>CBW</td>
<td>Current body weight (lbs)</td>
</tr>
<tr>
<td>CLO</td>
<td>Clothing correction factor</td>
</tr>
<tr>
<td>CU</td>
<td>Carbohydrates used by muscle (grams)</td>
</tr>
<tr>
<td>DGLU</td>
<td>Glucose used initially (grams)</td>
</tr>
<tr>
<td>DRIP</td>
<td>One-third maximum evaporative capacity (kcal/min)</td>
</tr>
<tr>
<td>DSW</td>
<td>Sweat rate as a function of skin temperature (kg/hr)</td>
</tr>
<tr>
<td>DTB</td>
<td>Body temperature change (Deg. C/6 sec)</td>
</tr>
<tr>
<td>EMAX</td>
<td>Maximum evaporative capacity (kcal/6 sec)</td>
</tr>
<tr>
<td>EXDESC</td>
<td>Type of exercise</td>
</tr>
<tr>
<td>FRAME</td>
<td>Body frame size of user (small/medium/large)</td>
</tr>
<tr>
<td>FT</td>
<td>Height of user (inches)</td>
</tr>
<tr>
<td>FTSC</td>
<td>Skin temperature (Deg. F)</td>
</tr>
<tr>
<td>GAIN1</td>
<td>Gain constant used in modifying sweat rate based on level of training</td>
</tr>
<tr>
<td>GAIN2</td>
<td>Gain constant used in modifying sweat rate based on level of dehydration</td>
</tr>
<tr>
<td>GENDER</td>
<td>Gender of the user (male/female)</td>
</tr>
<tr>
<td>GLU</td>
<td>Glucose used (grams/6 sec)</td>
</tr>
<tr>
<td>HC</td>
<td>Convective heat transfer coef. (kcal/(m^2<em>hr</em>C))</td>
</tr>
<tr>
<td>HR</td>
<td>Radiative heat transfer coef. (kcal/(m^2<em>hr</em>C))</td>
</tr>
<tr>
<td>HD</td>
<td>Difference between heat generated and heat lost</td>
</tr>
<tr>
<td>HT</td>
<td>Height of user (cm)</td>
</tr>
<tr>
<td>HV</td>
<td>Heat transfer coefficient for vaporization (kcal/(m^2<em>hr</em>mmHg))</td>
</tr>
<tr>
<td>HVAP</td>
<td>Heat of vaporization of water @ 37 Deg. C (kcal/mol)</td>
</tr>
<tr>
<td>ICLO</td>
<td>Input indication of user clothing factor</td>
</tr>
<tr>
<td>IOPTION</td>
<td>Program calculation option</td>
</tr>
<tr>
<td>KG</td>
<td>Mass transfer coefficient for evaporation of water (kg/(hr<em>mmHg</em>m^2)))</td>
</tr>
<tr>
<td>MAXSR</td>
<td>Maximum sweat rate depending on level of acclimatization (kg/hr)</td>
</tr>
<tr>
<td>MDOT</td>
<td>Metabolic energy rate (kcal/hr)</td>
</tr>
<tr>
<td>MILE</td>
<td>Time of mile (min)</td>
</tr>
<tr>
<td>MIN</td>
<td>Time of mile (min)</td>
</tr>
</tbody>
</table>
* MKG  Athlete's mass (kg)
* N    No, answer to question
* NAME Name of program user
* NUMDAT  Counter used to determine number of data points sent
to output file
* PA  Vapor pressure of water at ambient temperature
   using the Antoine equation (mmHg)
* PS  Vapor pressure of water at skin temperature
   using the Antoine equation (mmHg)
* PVO Percent maximum oxygen capacity required for a
certain level of exercise (%)
* QCONV  Heat transfer by convection (kcal/min)
* QEVAP  Heat transfer by evaporation (kcal/min)
* QRAD  Heat transfer by radiation (kcal/min)
* QRES  Energy lost in respiration (kcal/min)
* QTOT  Total heat transfer (kcal/min)
* QUAD  Glycogen content in quad (mmol)
* RHUM Relative humidity
* SEC  Time of mile (s)
* SGLU Rate of glucose consumption at a certain level
   of exercise (mmol/kg*min)
* SKO  Thermal conductivity of skin at given body core
   temperature (kcal/(m*min*Deg. C))
* SWEAT DSWEAT conversion (kg/6 sec)
* TBC  Body core temperature (Deg. C)
* TIC  Initial ambient temperature (Deg. C)
* TIF  Initial ambient temperature (Deg. F)
* TRICEP Glycogen content in tricep (mmol)
* TSC  Relation for body skin temperature
* TSWEAT Added water loss (kg)
* TT  Time counter (min)
* VAIRL Rate of air intake for a particular exercise
   wrt MDOT (l/min)
* VMETPS Velocity created by the athlete (m/s)
* VMPS  Velocity created by the athlete (miles/hr)
* VELTOT  Total net velocity of the wind (created by the
   athlete and the environmental conditions)
* VOL  Required oxygen consumption for a particular
   exercise
* VOMAX  Variable for maximum oxygen capacity of a particular
   athlete
* WMETPS Wind velocity (m/s)
* WMPH  Wind velocity (miles/hr)
* WTLBS  Athlete's weight (lbs)
* WW  Response to wind direction question
* Y  Yes, answer to question
* YA  Mole fraction of water in air
* Z  Rate of mass transfer by evaporation (kg/hr)

**************************************************************************************
SECTION 1: This section asks the user to input their physical characteristics, environmental conditions, and the desired type of exercise.

--- Enter physical characteristics

C

write(*,2) 'Please enter your name (50 character max): >'
read(*,4) NAME

write(*,2) 'Gender is used in calculating basal metabolic rate.'
write(*,1)
write(*,*) 'If you are a female, please enter F;
write(*,*) 'if you are a male, please enter M: >'
read (*,4) GENDER

if ((GENDER .NE. 'F') .AND. (GENDER .NE. 'M'))
   .AND. (GENDER .NE. 'f') .AND. (GENDER .NE. 'm')) then
   write(*,5)
   GOTO 9
end if

write(*,2) 'This program is designed for athletes between the ages of 18 and 75.'
write(*,*) 'Please enter your age in years: >'
read(*,*) AGE

if ((AGE .LT. 18) .OR. (AGE .GT. 75)) then
   write(*,5)
   GOTO 10
end if

write(*,2) 'Please enter your height (feet, inches): >'
read(*,*) FT, IN

if ((FT .LT. 0). OR. (IN .LT. 0) .OR. (IN .GT. 12)
+ .OR. FT .GT. 7) then
   write(*,5)
   GOTO 11
end if

write(*,2) 'Please enter your body weight (60 - 300 lbs): >'
read(*,*) WTLBS

if ((WTLBS .LT. 60) .OR. (WTLBS .GT. 300)) then
   write(*,5)
GOTO 12
end if

13 write(*,2) 'Please enter your approximate muscular build: >'
write(*,*) '(small, medium, large)'
read(*,4) FRAME

if ((FRAME .NE. 'small') .AND. (FRAME .NE. 'large')
+ .AND. (FRAME .NE. 'medium')) then
  write(*,5)
  GOTO 13
end if

14 write(*,2) 'Enter the number which best corresponds to your'
write(*,*) 'clothing (in addition to your shoes): >'
write(*,1)
write(*,*) 'Shorts only, enter 1.'
write(*,*) 'Shorts and short-sleeve shirt or singlet, enter 2.'
write(*,*) 'Shorts and long-sleeve shirt, enter 3.'
write(*,*) 'Legs covered and long-sleeve shirt, enter 4.'
write(*,*) 'Legs, arms, hands, and head all covered, enter 5.'
read(*,*) ICLO

if (ICLO .NE. 1 .AND. ICLO .NE. 2 .AND. ICLO .NE. 3
+ .AND. ICLO .NE. 4 .AND. ICLO .NE. 5) then
  write(*,5)
  GOTO 14
end if

C — Environmental conditions

15 write(*,2) 'Please enter the air temperature (-40 to 120 Deg. F): >'
read(*,*) TIF

if ((TIF .LT. -40.0) .OR. (TIF .GT. 120)) then
  write(*,5)
  GOTO 15
end if

16 write(*,2) 'Please enter wind velocity (0 - 50 miles/hr): >'
read(*,*) WMPH

if ((WMPH .GT. 50.0) .OR. (WMPH .LT. 0.0)) then
  write(*,5)
  GOTO 16
end if

if (WMPH .EQ. 0.0) then
  WMPH = WMPH + 1.0 ! Add 1 mph to account for free convection
  GOTO 17
end if
write(*,1) 'On the average, is the wind opposing you (A) or'
write(*,*) 'following you (F), or neither (N),'
write(*,*) 'Please enter either A, F, or N: >'
read(*,4) WW

if ((WW .NE. 'A') .AND. (WW .NE. 'a') .AND. (WW .NE. 'F') .AND. (WW .NE. 'f') .AND. (WW .NE. 'N') .AND. (WW .NE. 'n')) then
write(*,5)
GOTO 25
end if

write(*,2) 'Please enter the relative humidity as a decimal: >'
read(*,*) RHUM

if ((RHum .LT. 0.0) .OR. (RHum .GT. 1.0)) then
write(*,5)
GOTO 17
end if

-- Enter activity information (duration,level,type)
write(*,2) 'Would you like
the
program to:
write(*,3) '(1) recommend a duration of exercise?
write(*,3) '(2) allow you to choose a duration of exercise?
write(*,1)
write(*,*) 'Please enter I or 2: >'
read(*,*) IOPTION

if (IOPTION .NE. 1 .AND. IOPTION .NE. 2) then
write(*,5)
GOTO 18
end if

if (IOPTION .EQ. 1) then
TTMAX = 300
else
write(*,2) 'Please enter the duration of exercise (1 - 300 min): >'
read(*,*) TTMAX
if ((TTMAX .LE. 0) .OR. (TTMAX .GT. 300)) then
write(*,5)
GOTO 19
end if
end if

if (TT .GT. 15) then
! Bypass activity information that
goto 41
! does not change during subsequent
tests of exercise conditions.
end if

-- The user's time for the mile run provides information about
C
maximal oxygen capacity and therefore maximal level of exercise
write(*,2) 'Do you know your best recent time for the mile?'
write(*,*) 'If YES, please enter Y; if NO, please enter N: >'
read(*,4) ANS
if ((ANS .NE. 'Y') .AND. (ANS .NE. 'N') .AND. (ANS .NE. 'y') .AND. (ANS .NE. 'n')) then
    write(*,5)
    GOTO 20
end if
if ((ANS .EQ. 'Y') .OR. (ANS .EQ. 'y')) then
    write(*,2) 'Please enter your time for the mile (min,sec): >'
    read(*,*), MIN,SEC
    if ((MIN .LT. 4) .OR. (MIN .GT. 20) .OR. (SEC .GT. 60) .OR. (SEC .LT. 0)) then
        write(*,5)
        GOTO 21
    end if
    else
        MIN = 0
        SEC = 0
end if
write(*,2) 'If best time for the mile is unknown, then maximum oxygen capacity is based on age and activity level.'
write(*,2) 'Please select your normal level of activity:'
write(*,3) '1) Active (well trained in distance sports)'
write(*,3) '2) Inactive (no continuous training)'
write(*,1)
write(*,5) 'Please enter 1 or 2: >'
read(*,*) ACTLEV
if ((ACTLEV .NE. 1) .AND. (ACTLEV .NE. 2)) then
    write(*,5)
    GOTO 39
end if
write(*,2) 'Please select level of acclimatization:'
write(*,3) '1) Not acclimated to heat of cold'
write(*,3) '2) Heat acclimated:
    7-10 days training, 2-4 hrs/day, > 95 Deg. F'
write(*,3) '3) Cold acclimated'
write(*,1)
write(*,5) 'Please enter 1, 2, or 3: >'
read(*,*) ACCLEV
if ((ACCLEV .NE. 1) .AND. (ACCLEV .NE. 2) .AND. ACCLEV .NE. 3) then
    write(*,5)
    GOTO 40
end if

41  write(*,2) 'Please choose a type of exercise from the
+ list below:
write(*,1)
write(*,*) 'Walking, running, cycling, skiing, basketball'
write(*,*) 'boxing, soccer, rowing, tennis, circuit training,'
write(*,*) 'field hockey, football, squash, or aerobics.'
read(*,4) EXDESC

if ((EXDESC .NE. 'running') .AND.
+ (EXDESC .NE. 'walking') .AND.
+ (EXDESC .NE. 'cycling') .AND.
+ (EXDESC .NE. 'skiing') .AND.
+ (EXDESC .NE. 'basketball') .AND.
+ (EXDESC .NE. 'boxing') .AND.
+ (EXDESC .NE. 'soccer') .AND.
+ (EXDESC .NE. 'rowing') .AND.
+ (EXDESC .NE. 'tennis') .AND.
+ (EXDESC .NE. 'circuit training') .AND.
+ (EXDESC .NE. 'field hockey') .AND.
+ (EXDESC .NE. 'football') .AND.
+ (EXDESC .NE. 'squash') .AND.
+ (EXDESC .NE. 'aerobics')) then
  write(*,5)
  GOTO 41
end if

if ((EXDESC .EQ. 'walking') .OR. (EXDESC .EQ. 'running')) then
  write(*,2) 'Please enter your speed (1 - 16 miles/hr): >'
  read(*,*) VMPH
  if ((VMPH .LE. 0.0) .OR. (VMPH .GT. 16.0)) then
    write(*,5)
    GOTO 42
  end if
else if (EXDESC .EQ. 'cycling') then
  write(*,2) 'Please enter your speed (7 - 25 miles/hr): >'
  read(*,*) VMPH
  if ((VMPH .LT. 7.0) .OR. (VMPH .GT. 25.0)) then
    write(*,5)
    GOTO 43
  end if
else if (EXDESC .EQ. 'skiing') then
  write(*,2) 'Please enter your speed (2.5 - 8.8 miles/hr): >'
  read(*,*) VMPH
  if ((VMPH .LT. 2.5) .OR. (VMPH .GT. 8.8)) then
    write(*,5)
    GOTO 44
  end if
else if (EXDESC .EQ. 'boxing') then
  write(*,2) 'In a ring (ring) or sparring (spar)? >'

read(*,4) BOXANS
    if ((BOXANS .NE. 'ring') .AND. (BOXANS .NE. 'spar')) then
        write(*,5)
        GOTO 45
    end if
end if

*SECTION 2: This section defines/calculates constants and initial
*values based on user input.

CA = 0.006949
CB = 0.86
GLU = 0.0
HVAP = 10.39
NUMDAT = 0
TSWEAT = 0.0

--- Calculate total height (cm); convert weight to kg, temperature
to Deg. C, wind velocity to m/s, and velocity created by user to m/s.

HT = (12*FT + IN)*2.54
MKG = WTLBS * 0.4536
TIC = (TIF - 32.0)/1.8
WMETPS = 0.447 * WMPH
VMETPS = 0.447 * VMPH

--- Calculate surface area of body used on various mechanisms of
heat transfer as a percentage of body surface area

BSA = 0.00718 * (MKG**0.425) * (HT**0.725)
AC = 0.94*BSA
AR = 0.78*BSA
AV = 0.15*BSA
PA = PSAT(TIC)
YA = RHUM * PA / 760.0
PSI = PSAT(40)

--- Calculate basal metabolism (kcal/day) based on gender,
age, weight, and height

if ((GENDER .EQ. 'F') .OR. (GENDER .EQ. 'f')) then
    BASAL = 655.1 + 9.563*MKG + 1.850*HT - 4.676*AGE
else if ((GENDER .EQ. 'M') .OR. (GENDER .EQ. 'm')) then
    BASAL = 66.5 + 13.75*MKG + 5.003*HT - 6.775*AGE
end if
BASAL = BASAL/(24.0 * 60.0)  ! Change units to kcal/min

C — Initial values for energy loss terms based on basal metabolism

QRES = 0.1*BASAL
QRAD = 0.6*BASAL
QEVP = 0.1*BASAL
QCONV = 0.2*BASAL

C — Calculate initial skin temperature based on environmental temperature and body core temperature

TSCO = TBC - (TBC - TIC)/2.0

C — Determine volume of the body in cubic meters taking body build into account

BCM = WTLBS * 0.07 / 154.0

if (FRAME .EQ. 'small') then
   BCMB = BCM * 0.75
else if (FRAME .EQ. 'medium') then
   BCMB = BCM * 1.00
else if (FRAME .EQ. 'large') then
   BCMB = BCM * 1.25
end if

C — Determine clothing factor

if (ICLO.EQ.1) then
   CLOTHES = 'Shorts only'
   CLO = 1.0
else if (ICLO.EQ.2) then
   CLOTHES = 'Shorts and short-sleeve shirt or singlet'
   CLO = 0.70
else if (ICLO.EQ.3) then
   CLOTHES = 'Shorts and long-sleeve shirt'
   CLO = 0.55
else if (ICLO.EQ.4) then
   CLOTHES = 'Legs covered and long-sleeve shirt'
   CLO = 0.30
else if (ICLO.EQ.5) then
   CLOTHES = 'Legs, arms, hands, and head all covered'
   CLO = 0.25
end if

C — Set gain in sweat rate equation based on level of training

if (ACTLEV .EQ. 1) then
   GAIN1 = 1.65
else
GAIN1 = 1.0
end if

--- Define initial body core temperature (deg. C) and maximum sweat rate (kg/hr) based on level of acclimatization

if ((ACCLEV .EQ. 2) .AND. (ACTLEV .EQ. 1)) then
   MAXSR = 4.0
   TBC = 36.7
else
   MAXSR = 1.5
   TBC = 37.0
end if

--- Calculate glycogen content in muscles in mmoles using typical values for muscle distribution in the body and glycogen density

QUAD = (0.7 * 0.665 * 0.1101 * BCMB * 17750 * 5.5)
HAM = (0.3 * 0.665 * 0.1101 * BCMB * 17750 * 5.5)
BICEP = (0.5 * 0.0241 * 0.70 * BCMB * 17750 * 5.5)
TRICEP = (0.5 * 0.70 * 0.0241 * BCMB * 17750 * 5.5)
CALF = (0.046 * 0.55 * 0.6 * BCMB * 17750 * 5.5)

--- Calculate metabolic rate for the type of exercise selected

if ((EXDESC .EQ. 'walking') .OR. (EXDESC .EQ. 'running')) then
   if ((VMPH .GT. 0.0) .AND. (VMPH .LT. 2.0)) then
      MDOT = 125.0
   else if ((VMPH .GE. 2.0) .AND. (VMPH .LT. 3.0)) then
      MDOT = 151.0 + (VMPH - 2.0) * 82.0
   else if ((VMPH .GE. 3.0) .AND. (VMPH .LT. 4.0)) then
      MDOT = 233.0 + (VMPH - 3.0) * 133.0
   else if ((VMPH .GE. 4.0) .AND. (VMPH .LT. 5.0)) then
      MDOT = 366.0 + (VMPH - 4.0) * 210.0
   else if ((VMPH .GE. 5.0) .AND. (VMPH .LT. 6.0)) then
      MDOT = 576.0 + (VMPH - 5.0) * 46.0
   else if ((VMPH .GE. 6.0) .AND. (VMPH .LT. 8.0)) then
      MDOT = 622.0 + (VMPH - 6.0) * 43.5
   else if ((VMPH .GE. 8.0) .AND. (VMPH .LT. 10.0)) then
      MDOT = 709.0 + (VMPH - 8.0) * 172.5
   else if ((VMPH .GE. 10.0) .AND. (VMPH .LT. 12.0)) then
      MDOT = 840.0 + (VMPH - 10.0) * 82.0
   else
      MDOT = 125.0
   end if
end if

MDOT = MDOT * WTLBS/154.0

if (WMPH .EQ. 1.0) then
   VELTOT = VMETPS
end if

if ((WW .EQ. 'A') .OR. (WW .EQ. 'a')) then
VELTOT = VMETPS + WMETPS
else if ((WW .EQ. 'F') .OR. (WW .EQ. 'f')) then
    VELTOT = ABS(VMETPS - WMETPS)
else if ((WW .EQ. 'N') .OR. (WW .EQ. 'n')) then
    VELTOT = VMETPS
end if

C  ---Calculate heat transfer coefficients for various mechanisms of C  heat transfer based on the clothing factor and velocity
HC = CLO*6.4*VELTOT**0.67
HR = 4.3*CLO
HV = 11.9*CLO*VELTOT**0.6
KG = 0.0206332 * CLO*VELTOT**0.6

C  --- Calculate rate of air intake (mole/hr), and rate of C  oxygen required for exercise (l/min)
VAIRL = 0.0264 * MDOT - 0.714
AIRMOL = VAIRL * 60.0 / (0.08206 * (TIC + 273.15))
VOL = 0.002768 * MDOT - 0.07164

C  --- Calculate maximal oxygen capacity for users who do not know their C  best recent time for the mile run. Calculation is based on user's C  activity, age, and weight.
if ((ANS .EQ. 'N') .OR. (ANS .EQ. 'n') .AND. ACTLEV .EQ. 1) then
    VOMAX = (59.943 - 0.346*AGE) * MKG/1000.0
else if ((ANS .EQ. 'N') .OR. (ANS .EQ. 'n') .AND. ACTLEV .EQ. 2) then
    VOMAX = (103.43 - 1.257*AGE) * MKG/1000.0
end if

MILE = MIN + SEC/60.0
VOMAX = (1608.0 + 30.0*MILE)/(5.0*(MILE + 1.0))*MKG/1000.0

C  --- Calculate percent maximum oxygen capacity required for C  level of exercise selected
PVO = VOL / VOMAX * 100.0
IPVO = PVO
write(*,7) 'You are exercising at ',IPVO,' % of your capacity.'
write(*,98)

C  --- Calculate rate of glucose or carbohydrate consumption C  at level of exercise selected
SGLU = 0.005827*PVO+0.00004545*(PVO**2)+0.000001667*(PVO**3)
if (ACTLEV .EQ. 1) SGLU = 0.75*SGLU
SECTION 3: This section calculates change in body temperature, weight loss, and carbohydrate usage with time. Data are written to data files every 5 minutes and printed to the screen every 15 minutes.

DO 150 TT = 15,300,15
   DO 111 j = 1,3
      DO 110 k = 0,4,9,0.1
         if (TT .EQ. 15 .AND. j .EQ. 1 .AND. k .EQ. 0) then
            TSC = TSCO
            TTSC = TSCO
         end if

C — Calculate thermal conductivity as function of body temperature to account for vasoconstriction (TBC < 37 Deg. C) or vasodilation (TBC > 37 Deg. C)

if (TBC .LT. 37.0) then
   SKO = 0.5*(1.0 - (37.0 - TBC)/6.0)
else
   SKO = 0.5*(1.0 + (TBC - 37.0)/3.0)
end if

C — Calculate new skin temperatures

TNSC = TBC - ((QRES + QEVAP + QCONV + QRAD)/10.0)/(1.8*SKO)

C — Average last three skin temperatures

TSC = (TSC + TTSC + TNSC)/3.0
TTSC = TNSC

TICL = TIC + 1.0
TBCL = TBC - 0.5

C — Skin temperature cannot be < 1 Deg. C more than environmental temperature

if (TSC .LT. TICL) TSC = TICL

C — Skin temperature cannot be > 1/2 Deg. C less than body core temperature

if (TSC .GT. TBCL) TSC = TBCL

PS = PSAT(TSC)
C — Calculate rate of evaporative mass transfer
   
   \[ Z = KG*AV*(PS - RHUM*PA) \]

C — If core temperature < 37 Deg. C water loss is determined by the rate of evaporation
   
   if (TBC .LT. 37.0) DSWEAT = Z

C — Determine level of dehydration to modify slope of DWEAT vs. TBC relationship used in determining sweat rate. The gains defined with respect to a given level of dehydration are NOT specifically provided in literature, but have been arbitrarily estimated using a hypothesized trend.

   if (TSWEAT .LT. 0.01*MKG) then
     GAIN2 = 1.0
   else if ((TSWEAT .GE. 0.01*MKG).AND.(TSWEAT .LT. 0.02*MKG)) then
     GAIN2 = 0.95
   else if ((TSWEAT .GE. 0.02*MKG).AND.(TSWEAT .LT. 0.03*MKG)) then
     GAIN2 = 0.85
   else if ((TSWEAT .GE. 0.03*MKG).AND.(TSWEAT .LT. 0.04*MKG)) then
     GAIN2 = 0.75
   else if ((TSWEAT .GE. 0.04*MKG).AND.(TSWEAT .LT. 0.05*MKG)) then
     GAIN2 = 0.65
   end if

C — Call subroutine to calculate sweat rate (water loss rate)
   
   DSWEAT = WATERLOSS(TSC,TBC,MDOT,GAIN1,GAIN2)

   if (DSWEAT .GT. MAXSR) DSWEAT = MAXSR

   SWEAT = DSWEAT / 600.0  !Convert rate to kg/6 sec

C — Calculate energy lost through respiration, evaporation, convection and radiation

   PEXP = PSAT(TBC)
   YEXP = PEXP/760.0

   QRES = AIRMOL*(CA*(TBC - TIC) + HVAP*(YEXP - YA))/60.0
   QEVP = HV*AV*(PS - RHUM*PA)/60.0
   QCONV = HC*AC*(TSC - TIC)/60.0
   QRAD = HR*AR*(TSC - TIC)/60.0
   QTOT = QRES + QEVP + QCONV + QRAD - MDOT/60.0

   EMAX = QEVP*0.1  ! Convert rate to kcal/6 sec
   DRIP = EMAX/3.0  ! Determine point at which run-off occurs (1/3 max evaporative capacity)

C — Calculate body temperature change in 6 seconds by using
C dividing new rate of heat loss and new body weight

\[ DTB = \frac{-(QTOT/10.0))/((MKG - TSWEAT)*CB)}{}} \]

C — Calculate new body core temperature

\[ TBC = TBC + DTB \]

C — Add water loss in current period to the previous water loss.

C If the sweat rate is greater than or equal to 1/3 maximum evaporative
C capacity then loss of sweat can occur by dripping. If the sweat rate
C is less than 1/3 EMAX then loss of water occurs by evaporation.

if (SWEAT .GE. DRIP) then
    TSWEAT = TSWEAT + SWEAT
else
    TSWEAT = TSWEAT + Z/600.0
end if

if (j .eq. 1) then
   TS = k+(TT-15.0)
else if (j .eq. 2) then
   TS = (5.0 + k)+(TT-15.0)
else if (j .eq. 3) then
   TS = (10.0 + k)+(TT-15.0)
end if

C — Calculate the amount of glucose used in period

\[ DGLU = SGLU * \frac{1.030}{10.0} \]

C — Calculate total glucose usage

\[ GLU = GLU + DGLU \]

C — Determine whether maximum glucose usage has been surpassed

if ((EXDESC .EQ. 'running') .OR. (EXDESC .EQ. 'walking')) then
    if (GLU .GE. CALF) then
        write(*,100)
        write(*,101)
        GOTO 275
    end if
    else if (EXDESC .EQ. 'cycling') then
        if ((GLU .GE. (0.7 * QUAD)) .OR. (GLU .GE. (0.3 * (BICEP + TRICEP))))) then
            write(*,100)
            write(*,101)
            GOTO 275
        end if
    else if (EXDESC .EQ. 'rowing') then

if (GLU .GE. (0.5 * QUAD))
+ .OR. (GLU .GE. (0.5 * BICEP))) then
    write(*,100)
    write(*,101)
    GOTO 275
end if
else if (EXDESC .EQ. 'circuit training') then
    if ((GLU .GE. (0.2 * QUAD)) .OR. (GLU .GE. (0.2 * BICEP))
+ .OR. (GLU .GE. (0.2 * HAM)) .OR. (GLU .GE. (0.2 * TRICEP))
+ .OR. (GLU .GE. (0.2 * CALF))) then
        write(*,100)
        write(*,101)
        GOTO 275
    end if
else if (.NOT. ((EXDESC .EQ. 'running') .OR.
+ (EXDESC .EQ. 'walking') .OR. (EXDESC .EQ. 'cycling')
+ .OR. (EXDESC .EQ. 'rowing')
+ .OR. (EXDESC .EQ. 'circuit training'))) then
    if (GLU .GE. 92.7) then
        write(*,100)
        write(*,101)
        GOTO 275
    end if
end if

C — Determine if hypothermia has been achieved

if (TBC .LE. 34.0) then
    write(*,1) 'HYPOTHERMIA!!!'
    write(*,*) 'Body core temperature has fallen below 92 Deg. F.'
    GOTO 275
end if

C — Determine if dehydration limit has been exceeded

if (TSWEAT .GE. 0.05*MKG) then
    write(*,1) 'DEHYDRATION!!!'
    write(*,*) 'Loss of body water exceeds 5% total body weight.'
    GOTO 275
end if

C — Determine if exercise duration has been exceeded
if (TT .GT. TTMAX) then
    write(*,1) 'Exercise duration is complete!'
    GOTO 275
end if

T = TS + 0.1

C CONTINUE

C --- Write data to output file

FTSC = 1.8 * TSC + 32.0
CBT = 1.8 * TBC + 32.0
HD = -QTOT
CBW = (MKG - TSWEAT) * 2.2046
CU = GLU * 100.0 / 92.7

open (unit = 1, type = 'new', name = 'BODYTEMP.DAT')
open (unit = 2, type = 'new', name = 'QTOT.DAT')
open (unit = 3, type = 'new', name = 'BODYWT.DAT')
open (unit = 4, type = 'new', name = 'SKINTEMP.DAT')

write(1,99) T, CBT
write(2,99) T, HD
write(3,99) T, CBW
write(4,99) T, FTSC

C --- Count number of data pts.

NUMDAT = NUMDAT + 1

CONTINUE

C --- Print values to the screen every 15 min.

write(*,6) 'AFTER ', T, ' MINUTES:
write(*,1)
write(*,202) 'Rate of heat loss by respiration = ', QRES, ' kcal/min'
write(*,202) 'Rate of heat loss by evaporation = ', QEVAP, ' kcal/min'
write(*,202) 'Rate of heat loss by convection = ', QCONV, ' kcal/min'
write(*,202) 'Rate of heat loss by radiation = ', QRAD, ' kcal/min'
write(*,202) 'Heat accumulation in body = ', HD, ' kcal/min'
write(*,1)
write(*,201) 'Skin temperature = ', FTSC, ' Deg. F'
write(*,201) 'Current body temperature = ', CBT, ' Deg. F'
write(*,201) 'Current body weight = ', CBW, ' lbs'
read(*,*)

C --- Continue simulation if hyperthermia, hypothermia, dehydration or
exhaustion have not been reached
CONTINUE

write(*,7) 'Total exercise duration = ',T,' min'
write(*,269)
read(*,*)

***
#*******************
_.,I. iI. t. _.,I. _.* _.******************* _.** _.******* _ _ _*** ***** *** _ _ _******* _

* SECTION 4: This section calls subroutines FUELUSE, PRINTAB, PLOTDATA to estimate fuel usage, print a summary and plot data from the simulation, respectively.

*****************************************************************************

CALL FUELUSE(TT,VOL,PVO,TOTCAL,CHCAL,LIPCAL,CHUSED,LIPUSED)

CALL PRINTAB

write(*,2) 'Would you like to plot any of the following results from the simulation run?'
write(*,1)
write(*,3) '1) Body Temperature vs. Time'
write(*,3) '2) Heat Loss vs. Time'
write(*,3) '3) Body Weight vs. Time'
write(*,3) '4) Skin Temperature vs. Time'
write(*,1)
write(*,*) 'If YES, please enter Y; if NO, please enter N: >'
read(*,4) ANS

if ((ANS .NE. 'Y') .AND. (ANS .NE. 'N') .AND. (ANS .NE. 'y') .AND. (ANS .NE. 'n')) then
   write(*,5)
   GOTO 285
end if

if ((ANS .EQ. 'Y') .OR. (ANS .EQ. 'y')) then
   call PLOTDATA(NUMDAT)
else
   GOTO 300
end if

write(*,2) 'Would you like to test some other conditions?'
write(*,*) 'If YES, please enter Y; if NO, please enter N: >'
READ (*,4) ICASE

if ((ICASE .NE. 'Y') .AND. (ICASE .NE. 'y') .AND. (ICASE .NE. 'N') .AND. (ICASE .NE. 'n')) then
   write(*,5)
   GOTO 301

else
   GOTO 300
end if
else if ((ICASE .EQ. 'Y') .OR. (ICASE .EQ. 'y')) then
    close (1)
    close (2)
    close (3)
    close (4)
    GOTO 14
end if

1 format(/,1X,A)
2 format(/,1X,A)
3 format(1X,TR5,A)
4 format(A)
5 format(/,1X,'You have entered an invalid response,
+ please try again.',/)
6 format (/,1X,A,F5.1,A)
7 format (/,1X,A,F5.1,A)
98 format(1X,60('-'),/)
99 format (1X,F5.1,F10.2)
100 format(1X,'EXHAUSTION!!!')
101 format(1X,'You have depleted the carbohydrate stores in the
+ working muscle(s).')
201 format (1X,A,F7.1,A)
202 format (1X,A,F7.2,A)
269 format(/,1X,TR20,'Please press <RETURN> to continue.',///)

STOP
END

**************************************************************************************
* This section contains FUNCTION and SUBROUTINE programs to print a
* summary table, plot the data, calculate saturated water vapor
* pressure, sweat rates, and fuel use.
*
* SUBROUTINE PRINTTAB prints a table of results after the simulation is complete.
**************************************************************************************

SUBROUTINE PRINTTAB

real WTLBS, MKG, RHUM, MDOT, VMPH, TIF, CBW, CBT
real TOTCAL, CHCAL, LIPCAL, LIPUSED, CHUSED, FTSC, VOL

character NAME*50, EXDESC*50, CLOTHES*50
integer AGE, FT, IN, MIN, SEC, TT, ACTLEV

common NAME, FT, IN, WTLBS, AGE, CLOTHES, MIN, SEC, ACTLEV, EXDESC
common MDOT, VOL, VMPH, TIF, RHUM, WMPH, TS, CBW, CBT, FTSC, TOTCAL
common CHCAL,LIPCAL,CHUSED,LIPUSED

write(*,9)
write(*,*) 'SUMMARY OF EXERCISE SIMULATION FOR: >'
write(*,*) NAME

write(*,2) '— Characteristics of the Athlete'
write(*,6) 'Height: ',FT,' ft. ',IN,' inches'
write(*,4) 'Weight: ',WTLBS,' lbs.'
write(*,6) 'Age: ',AGE,' years'
write(*,*) 'Clothing: ',CLOTHES
write(*,1)
write(*,6) 'Best recent time for the mile: ',MIN,' min ',SEC,' sec'
write(*,*) 'Where: 0 - not available'
write(*,1)
write(*,8) 'Activity level = ',ACTLEV
write(*,*) 'Where: 1 - Active, 2 - Inactive'
write(*,10)
read(*,*)

write(*,2) '— Exercise Data'
write(*,3) EXDESC
write(*,1)
write(*,4) 'Metabolic rate = ',MDOT,' kcal/hr'
write(*,4) 'Oxygen Consumption = ',VOL,' liters oxygen/min'
write(*,4) 'Exercise Velocity = ',VMPH,' miles/hr (0 = N/A)'
write(*,2) '— Environmental Conditions'
write(*,4) 'Air temperature = ',TIF,' Deg. F'
write(*,5) 'Relative humidity = ',RHUM
write(*,4) 'Wind velocity = ',WMPH,' miles/hr'
write(*,10)
read(*,*)

write(*,2) '— Final Simulation Values'
write(*,7) 'Duration of Exercise = ',TS,' min'
write(*,4) 'Final body weight = ',CBW,' lbs'
write(*,4) 'Final body temp. = ',CBT,' Deg. F'
write(*,4) 'Final skin temp. = ',FTSC,' Deg. F'

write(*,2) '— Fuel Use'
write(*,4) 'Total energy spent = ',TOTCAL,' kcal'
write(*,4) ' Carbohydrates = ',CHCAL,' kcal'
write(*,4) ' Fats = ',L_L,' kcal'
write(*,1)
write(*,4) 'Carbohydrates used: ',CHUSED,' grams'
write(*,4) 'Fats used: ',LIPUSED,' grams'
write(*,10)
read(*,*)

1 format(1X,/)
SUBROUTINE PLOTDATA(NUMDAT)

        real LINE(0:100),STAR,SCALE,X(100),XMAX,XMIN
        real Y(100),YMAX,YMIN
        integer CHOICE,NUMDAT,PF
        character FILENAME*15,PRINTFILE*15,YVAR*25,XVAR*15

        write(*,1)
        read(*,*) CHOICE

        if (CHOICE .LE. 4 .AND. CHOICE .GE. 1) then
          XVAR = 'TIME (min)'
        else
          write(*,2)
          GOTO 290
        end if

        if (CHOICE .EQ. 1) then
          FILENAME = 'BODYTEMP.DAT'
          PRINTFILE = 'BODYTEMP.TXT'
          PF = 5
          YVAR = 'BODY TEMP. (Deg. F)'
        else if (CHOICE .EQ. 2) then
          FILENAME = 'QTOT.DAT'
          PRINTFILE = 'QTOT.TXT'
          PF = 6
          YVAR = 'HEAT LOSS (kcal/min)'
        else if (CHOICE .EQ. 3) then
          FILENAME = 'BODYWT.DAT'
          PRINTFILE = 'BODYWT.TXT'
          PF = 7
          YVAR = 'BODY WEIGHT (lbs)'
        else if (CHOICE .EQ. 4) then
          FILENAME = 'SKINTEMP.DAT'
        END
PRINTFILE = 'SKINTEMP.TXT'
PF = 8
YVAR = 'SKIN TEMP. (Deg. F)'
end if

C -- Input data

open (unit = CHOICE, status = 'old', file = FILENAME)
rewind (CHOICE)
open (unit = PF, status = 'new', file = PRINTFILE)
rewind (PF)

do 6 i = 1, NUMDAT
   read (CHOICE,12) X(i), Y(i)
6 continue

YMAX = Y(1)
YMIN = Y(1)

do 9 i = 1, NUMDAT
   if (Y(i).LT. YMIN) YMIN = Y(i)
   if (Y(i).GT. YMAX) YMAX = Y(i)
9 continue

SCALE = YMAX - YMIN

do 10 i = 0,55
   LINE(i) = ''
10 continue

C -- Plot data

write(*,3) PRINTFILE
write(PF,4) 'X','Y'
write(*,4) 'X','Y'

do k = 1, NUMDAT
   STAR = (Y(k) - YMIN)*55.0/SCALE
   LINE(STAR) = '*'
   write(PF,5) X(k),Y(k),(LINE(i),i = 0,55)
   write(*,5) X(k),Y(k),(LINE(i),i = 0,55)
   LINE(STAR) = ''
end do

write(PF,13)
write(*,13)
write(PF,14) XVAR,YVAR
write(*,14) XVAR,YVAR

write(*,*) 'Would you like to plot some other data?'
295 write(*,*) 'If YES, please enter Y; if NO, please enter N: ' >
READ (*,16) ICASE

if ((ICASE .NE. 'Y') .AND. (ICASE .NE. 'y') .AND. (ICASE .NE. 'N') .AND. (ICASE .NE. 'n')) then
  write(*,5)
  GOTO 295
end if

if ((ICASE .EQ. 'N') .OR. (ICASE .EQ. 'n')) then
  RETURN
else
  close (PF)
  write(*,11) '1) Body Temperature vs. Time'
  write(*,15) '2) Heat Loss vs. Time'
  write(*,15) '3) Body Weight vs. Time'
  write(*,15) '4) Skin Temperature vs. Time'
  GOTO 290
end if

1 format (/,1X,'Please enter 1, 2, 3, or 4: >')
2 format (1X,'You have entered an invalid response, + please try again: ',/)
3 format (/,1X,'The output file will be named ',A,/)  
4 format (3X,A1,TR7,A1,TR8,56('-'))
5 format (1X,F5.1,F10.2,TR3,'I',56A1,'I')
11 format (/,1X,A)
12 format (1X,F5.1,F10.2)
13 format (1X,TR19,56('-'),/)
14 format (1X,'Where: ',X='A10,4X,Y='A20,/)  
15 format (1X,A)
16 format (A)

RETURN
END

FUNCTION PSAT calculates saturated vapor pressure (mmHg) of water at given temperatures using the Antoine equation.

FUNCTION PSAT(T)

real T,A,B,C

C — Antoine constants for water

A = 8.10765
B = 1750.286
C = 235.0

PSAT = 10**((A - B/(T+C))
FUNCTION WATERLOSS(TSC,TBC,MDOT,GAIN1,GAIN2)

real TBC,TSC,MDOT,GAIN1,GAIN2

GAIN = GAIN1*GAIN2

if (MDOT .LE. 300.0) then
    DSWEAT = 0.046122*GAIN*TBC - 1.16399
else if (MDOT .LE. 500.0 .AND. MDOT .GT. 300.0) then
    if (TSC .LE. 33.0) then
        DSWEAT = 0.063091*GAIN*TBC - 1.53415
    else
        DSWEAT = 0.295611*GAIN*TBC - 9.53531
    end if
else if (MDOT .LE. 700.0 .AND. MDOT .GT. 500.0) then
    if (TSC .LE. 33.4) then
        DSWEAT = 0.091916*GAIN*TBC - 2.21063
    else
        DSWEAT = 0.12109*GAIN*TBC - 16.9846
    end if
else if (MDOT .LE. 900.0 .AND. MDOT .GT. 700.0) then
    if (TSC .LE. 32.0) then
        DSWEAT = 0.223172*GAIN*TBC - 6.03852
    else
        DSWEAT = 0.390976*GAIN*TBC - 11.8576
    end if
else if (MDOT .LE. 1100.0 .AND. MDOT .GT. 900.0) then
    DSWEAT = 0.256577*GAIN*TBC - 5.75055
else if (MDOT .LE. 1300.0 .AND. MDOT .GT. 1100.0) then
    if (TSC .LE. 29.0) then
        DSWEAT = 0.439115*GAIN*TBC - 12.3871
    else
        DSWEAT = 0.314289*GAIN*TBC - 8.09200
    end if
else if (MDOT .LE. 1500.0 .AND. MDOT .GT. 1300.0) then
    if (TSC .LE. 28.6) then
        DSWEAT = 0.69908*GAIN*TBC - 21.2381
    else
        DSWEAT = 0.227225*GAIN*TBC - 4.99327
    end if
else if (MDOT .LE. 1700.0 .AND. MDOT .GT. 1500.0) then
    DSWEAT = 0.053090*GAIN*TBC + 1.639657
else if (MDOT .LE. 1900.0 .AND. MDOT .GT. 1700.0) then
    DSWEAT = ...
else if (MDOT .LE. 2100.0 .AND. MDOT .GT. 1900.0) then
    DSWEAT =...
else if (MDOT .LE. 2300.0 .AND. MDOT .GT. 2100.0) then
    DSWEAT =...
else if (MDOT .LE. 2500.0 .AND. MDOT .GT. 2300.0) then
    DSWEAT =...
else if (MDOT .LE. 2700.0 .AND. MDOT .GT. 2500.0) then
    DSWEAT =...
else if (MDOT .LE. 2900.0 .AND. MDOT .GT. 2700.0) then
    DSWEAT =...
else if (MDOT .LE. 3100.0 .AND. MDOT .GT. 2900.0) then
    DSWEAT =...
else if (MDOT .LE. 3300.0 .AND. MDOT .GT. 3100.0) then
    DSWEAT =...
else if (MDOT .LE. 3500.0 .AND. MDOT .GT. 3300.0) then
    DSWEAT =...
else if (MDOT .LE. 3700.0 .AND. MDOT .GT. 3500.0) then
    DSWEAT =...
else if (MDOT .LE. 3900.0 .AND. MDOT .GT. 3700.0) then
    DSWEAT =...
else if (MDOT .LE. 4100.0 .AND. MDOT .GT. 3900.0) then
    DSWEAT =...
else if (MDOT .LE. 4300.0 .AND. MDOT .GT. 4100.0) then
    DSWEAT =...
else if (MDOT .LE. 4500.0 .AND. MDOT .GT. 4300.0) then
    DSWEAT =...
else if (MDOT .LE. 4700.0 .AND. MDOT .GT. 4500.0) then
    DSWEAT =...
else if (MDOT .LE. 4900.0 .AND. MDOT .GT. 4700.0) then
    DSWEAT =...
else if (MDOT .LE. 5100.0 .AND. MDOT .GT. 4900.0) then
    DSWEAT =...
else if (MDOT .LE. 5300.0 .AND. MDOT .GT. 5100.0) then
    DSWEAT =...
else if (MDOT .LE. 5500.0 .AND. MDOT .GT. 5300.0) then
    DSWEAT =...
else if (MDOT .LE. 5700.0 .AND. MDOT .GT. 5500.0) then
    DSWEAT =...
else if (MDOT .LE. 5900.0 .AND. MDOT .GT. 5700.0) then
    DSWEAT =...
else if (MDOT .LE. 6100.0 .AND. MDOT .GT. 5900.0) then
    DSWEAT =...
else if (MDOT .LE. 6300.0 .AND. MDOT .GT. 6100.0) then
    DSWEAT =...
else if (MDOT .LE. 6500.0 .AND. MDOT .GT. 6300.0) then
    DSWEAT =...
else if (MDOT .LE. 6700.0 .AND. MDOT .GT. 6500.0) then
    DSWEAT =...
else if (MDOT .LE. 6900.0 .AND. MDOT .GT. 6700.0) then
    DSWEAT =...
else if (MDOT .LE. 7100.0 .AND. MDOT .GT. 6900.0) then
    DSWEAT =...
else if (MDOT .LE. 7300.0 .AND. MDOT .GT. 7100.0) then
    DSWEAT =...
else if (MDOT .LE. 7500.0 .AND. MDOT .GT. 7300.0) then
    DSWEAT =...
else if (MDOT .LE. 7700.0 .AND. MDOT .GT. 7500.0) then
    DSWEAT =...
else if (MDOT .LE. 7900.0 .AND. MDOT .GT. 7700.0) then
    DSWEAT =...
else if (MDOT .LE. 8100.0 .AND. MDOT .GT. 7900.0) then
    DSWEAT =...
else if (MDOT .LE. 8300.0 .AND. MDOT .GT. 8100.0) then
    DSWEAT =...
else if (MDOT .LE. 8500.0 .AND. MDOT .GT. 8300.0) then
    DSWEAT =...
else if (MDOT .LE. 8700.0 .AND. MDOT .GT. 8500.0) then
    DSWEAT =...
else if (MDOT .LE. 8900.0 .AND. MDOT .GT. 8700.0) then
    DSWEAT =...
else if (MDOT .LE. 9100.0 .AND. MDOT .GT. 8900.0) then
    DSWEAT =...
else if (MDOT .LE. 9300.0 .AND. MDOT .GT. 9100.0) then
    DSWEAT =...
else if (MDOT .LE. 9500.0 .AND. MDOT .GT. 9300.0) then
    DSWEAT =...
else if (MDOT .LE. 9700.0 .AND. MDOT .GT. 9500.0) then
    DSWEAT =...
else if (MDOT .LE. 9900.0 .AND. MDOT .GT. 9700.0) then
    DSWEAT =...
else if (MDOT .LE. 10100.0 .AND. MDOT .GT. 9900.0) then
    DSWEAT =...
else
    DSWEAT = 0.
end if
end if

WATERLOSS = DSWEAT

RETURN
END

SUBROUTINE FUELUSE(TT, VOL, PVO, TOTCAL, CHCAL, LIPCAL, CHUSED, LIPUSED)

real R, CALEQ, TOTVOL, TOTCAL, CHFRAC, CHCAL, PVO
real VOL, LIPCAL, CHEQ, LIPEQ, CHUSED, LIPUSED
integer TT

SUBROUTINE VARIABLES

VARIABLE DESCRIPTION
CALEQ Caloric equivalent of 1 liter of oxygen consumed
CHCAL Calories derived from carbohydrates
CHEQ Equivalent carbohydrates oxidized (grams) per 1 liter oxygen consumed
CHFRAC Percentage calories derived from carbohydrates
CHUSED Total carbohydrate oxidized (grams)
LIPCAL Calories derived from lipids
LIPEQ Equivalent lipids oxidized (grams) per 1 liter oxygen consumed
LIPUSED Total lipids oxidized (grams)
PVO Athlete's percent maximal oxygen capacity at R Nonproteic respiratory quotient as function of PVO and duration of exercise
TOTVOL Total volume of oxygen consumed
TOTCAL Total kcal utilized
TT Duration of exercise (min)
VOL Oxygen consumption at given metabolic rate exercise level selected

C Determine the nonproteic respiratory quotient equation for the duration of exercise. The equations for R are correspond to the following durations:
C 0 min. (ex. sprinters), 30 min., 60 min., 120 min., 180 min., 240 min. and 300 min. For durations of exercise other than these values, R is
C approximated using the equation defined at the nearest duration.

if (TT .LE. 2)                 R = 0.0021*PVO + 0.765
if (TT .GT. 2 .AND. TT .LE. 45) R = 0.0021*PVO + 0.753
if (TT .GT. 45 .AND. TT .LE. 90) R = 0.0022*PVO + 0.737
if (TT .GT. 90 .AND. TT .LE. 150) R = 0.0020*PVO + 0.740
if (TT .GT. 150 .AND. TT .LE. 210) R = 0.0019*PVO + 0.735
if (TT .GT. 210 .AND. TT .LE. 270) R = 0.0020*PVO + 0.713
if (TT .GT. 270)                 R = 0.0021*PVO + 0.769

if (TT .GT. 300) then
  write(*,1) 'Maximum duration limit is 5 hours.'
  write(*,*), 'Percentages of carbohydrates and fats used will be'
  write(*,2) 'based on this duration of exercise.'
end if

C — Calculate total energy expenditure.

CALEQ = 3.813 + 1.233*R
TOTVOL = TT*VOL
TOTCAL = TOTVOL*CALEQ

C — Calculate calories derived from each fuel

CHFRAC = (-239.32 + 340.32*R)/100.0
CHCAL = CHFRAC*TOTCAL
LIPCAL = (1.0 - CHFRAC)*TOTCAL

C — Calculate grams of carbohydrate and lipids oxidized

CHEQ = -2.977 + 4.195*R
CHUSED = TOTVOL*CHEQ
LIPEQ = 1.722 - 1.717*R
LIPUSED = TOTVOL*LIPEQ

1 format(/,1X,A)
2 format(1X,A,/)
This program is designed to inform well trained and novice athletes how long they may exercise at a certain level before one of the following conditions occur:

a) hyperthermia: estimated core temperature surpasses 40 Deg. C (104 Deg. F)
b) dehydration: loss of body water exceeds 5% of body weight
c) estimated muscle glycogen supply exceeded
d) hypothermia: estimated core temperature drops below 34 Deg. C (92 Deg. F)

Please enter your name (50 character max): > Megan Scherb

Gender is used in calculating basal metabolic rate.

If you are a female, please enter F; if you are a male, please enter M: > f

This program is designed for athletes between the ages of 18 and 75.

Please enter your age in years: > 24

Please enter your height (feet, inches): > 5.0 4.0

Please enter your body weight (60 - 300 lbs): > 135.0

Please enter your approximate muscular build: > (small, medium, large) medium

Enter the number which best corresponds to your clothing (in addition to your shoes): >

Shorts only, enter 1.
Shorts and short-sleeve shirt or singlet, enter 2.
Shorts and long-sleeve shirt, enter 3.
Legs covered and long-sleeve shirt, enter 4.
Legs, arms, hands, and head all covered, enter 5.

Please enter the air temperature (-40 to 120 Deg. F): > 70.0
Please enter wind velocity (0 - 50 miles/hr): > 0.0

Please enter the relative humidity as a decimal: > 0.5

Would you like the program to:
   (1) recommend a duration of exercise?
   (2) allow you to choose a duration of exercise?

Please enter 1 or 2: > 1

Do you know your best recent time for the mile?
If YES, please enter Y; if NO, please enter N: > y

Please enter your time for the mile (min, sec): > 7.0 30.0

If best time for the mile is unknown, then maximum oxygen capacity is based on age and activity level.

Please select your normal level of activity:
   1) Active (well trained in distance sports)
   2) Inactive (no continuous training)

Please enter 1 or 2: > 1

Please select level of acclimatization:
   1) Not acclimated to heat of cold
   2) Heat acclimated:
        7-10 days training, 2-4 hrs/day, > 95 Deg. F
   3) Cold acclimated

Please enter 1, 2, or 3: > 2

Please choose a type of exercise from the list below:

Walking, running, cycling, skiing, basketball
boxing, soccer, rowing, tennis, circuit training,
field hockey, football, squash, or aerobics.
running

Please enter your speed (1 - 16 miles/hr): > 7.5

You are exercising at 60.4 % of your capacity.
AFTER 15.0 MINUTES:

Rate of heat loss by respiration = 0.40 kcal/min
Rate of heat loss by evaporation = 2.60 kcal/min
Rate of heat loss by convection = 4.03 kcal/min
Rate of heat loss by radiation = 1.00 kcal/min
Heat accumulation in body = 2.01 kcal/min

Skin temperature = 97.8 Deg. F
Current body temperature = 99.2 Deg. F
Current body weight = 134.9 lbs

Please press <RETURN> to continue.

AFTER 30.0 MINUTES:

Rate of heat loss by respiration = 0.42 kcal/min
Rate of heat loss by evaporation = 2.71 kcal/min
Rate of heat loss by convection = 4.19 kcal/min
Rate of heat loss by radiation = 1.04 kcal/min
Heat accumulation in body = 1.69 kcal/min

Skin temperature = 98.9 Deg. F
Current body temperature = 100.2 Deg. F
Current body weight = 134.7 lbs

Please press <RETURN> to continue.

AFTER 45.0 MINUTES:

Rate of heat loss by respiration = 0.43 kcal/min
Rate of heat loss by evaporation = 2.80 kcal/min
Rate of heat loss by convection = 4.32 kcal/min
Rate of heat loss by radiation = 1.07 kcal/min
Heat accumulation in body = 1.42 kcal/min

Skin temperature = 99.8 Deg. F
Current body temperature = 101.0 Deg. F
Current body weight = 134.5 lbs

Please press <RETURN> to continue.

AFTER 60.0 MINUTES:
Rate of heat loss by respiration = 0.44 kcal/min
Rate of heat loss by evaporation = 2.88 kcal/min
Rate of heat loss by convection = 4.42 kcal/min
Rate of heat loss by radiation = 1.10 kcal/min
Heat accumulation in body = 1.20 kcal/min

Skin temperature = 100.5 Deg. F
Current body temperature = 101.6 Deg. F
Current body weight = 134.4 lbs

Please press <RETURN> to continue.

AFTER 75.0 MINUTES:

Rate of heat loss by respiration = 0.45 kcal/min
Rate of heat loss by evaporation = 2.95 kcal/min
Rate of heat loss by convection = 4.51 kcal/min
Rate of heat loss by radiation = 1.12 kcal/min
Heat accumulation in body = 1.01 kcal/min

Skin temperature = 101.1 Deg. F
Current body temperature = 102.2 Deg. F
Current body weight = 134.2 lbs

Please press <RETURN> to continue.

AFTER 90.0 MINUTES:

Rate of heat loss by respiration = 0.46 kcal/min
Rate of heat loss by evaporation = 3.00 kcal/min
Rate of heat loss by convection = 4.59 kcal/min
Rate of heat loss by radiation = 1.14 kcal/min
Heat accumulation in body = 0.85 kcal/min

Skin temperature = 101.6 Deg. F
Current body temperature = 102.7 Deg. F
Current body weight = 134.0 lbs

Please press <RETURN> to continue.

AFTER 105.0 MINUTES:

Rate of heat loss by respiration = 0.46 kcal/min
Rate of heat loss by evaporation = 3.05 kcal/min
Rate of heat loss by convection = 4.65 kcal/min
Rate of heat loss by radiation = 1.15 kcal/min
Heat accumulation in body = 0.72 kcal/min
Skin temperature = 102.1 Deg. F
Current body temperature = 103.1 Deg. F
Current body weight = 133.9 lbs

Please press <RETURN> to continue.

AFTER 120.0 MINUTES:

Rate of heat loss by respiration = 0.47 kcal/min
Rate of heat loss by evaporation = 3.09 kcal/min
Rate of heat loss by convection = 4.71 kcal/min
Rate of heat loss by radiation = 1.17 kcal/min
Heat accumulation in body = 0.61 kcal/min

Skin temperature = 102.4 Deg. F
Current body temperature = 103.4 Deg. F
Current body weight = 133.7 lbs

Please press <RETURN> to continue.

EXHAUSTION!!!
You have depleted the carbohydrate stores in the working muscle(s).
Total exercise duration = 132.8 min

Please press <RETURN> to continue.

-----------------------------
SUMMARY OF EXERCISE SIMULATION FOR:  >
Megan Scherb

--- Characteristics of the Athlete
Height: 5.0 ft. 4.0 inches
Weight: 135.0 lbs.
Age: 24 years
Clothing: Shorts and short-sleeve shirt or singlet

Best recent time for the mile: 7.0 min 30.0 sec
Where: 0 - not available

Activity level = 1
Where: 1 - Active, 2 - Inactive

Please press <Return> to continue.
--- Exercise Data

running

Metabolic rate = 602.5 kcal/hr
Oxygen Consumption = 1.6 liters oxygen/min
Exercise Velocity = 7.5 miles/hr (0 = N/A)

--- Environmental Conditions

Air temperature = 70.0 Deg. F
Relative humidity = 0.50
Wind velocity = 1.0 miles/hr

Please press <Return> to continue.

--- Final Simulation Values

Duration of Exercise = 132.8 min
Final body weight = 133.6 lbs
Final body temp. = 103.6 Deg. F
Final skin temp. = 102.6 Deg. F

--- Fuel Use

Total energy spent = 1050.2 kcal
- Carbohydrates = 563.4 kcal
- Fats = 486.8 kcal

Carbohydrates used: 136.7 grams
Fats used: 52.6 grams

Please press <Return> to continue.

Would you like to plot any of the following results from the simulation run?

1) Body Temperature vs. Time
2) Heat Loss vs. Time
3) Body Weight vs. Time
4) Skin Temperature vs. Time

If YES, please enter Y; if NO, please enter N: > Y

Please enter 1, 2, 3, or 4: > 1

The output file will be named BODYTEMP.TXT

<table>
<thead>
<tr>
<th>X</th>
<th>Y</th>
</tr>
</thead>
<tbody>
<tr>
<td>5.0</td>
<td>98.49</td>
</tr>
<tr>
<td>10.0</td>
<td>98.86</td>
</tr>
<tr>
<td>15.0</td>
<td>99.22</td>
</tr>
</tbody>
</table>
Where: \( X = \text{TIME (min)} \quad Y = \text{BODY TEMP. (Deg. F)} \)

Would you like to plot some other data?
If YES, please enter Y; if NO, please enter N: >

Y

1) Body Temperature vs. Time
2) Heat Loss vs. Time
3) Body Weight vs. Time
4) Skin Temperature vs. Time

Please enter 1, 2, 3, or 4: > 2

The output file will be named QTOT.TXT
Where: X = TIME (min)    Y = HEAT LOSS (kcal/min)

Would you like to plot some other data?
If YES, please enter Y; if NO, please enter N:  > Y

1) Body Temperature vs. Time
2) Heat Loss vs. Time
3) Body Weight vs. Time
4) Skin Temperature vs. Time

Please enter 1, 2, 3, or 4:  > 3

The output file will be named BODYWT.TXT

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<th>Y</th>
</tr>
</thead>
<tbody>
<tr>
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</tr>
<tr>
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<td>25.0</td>
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<td>133.63</td>
</tr>
<tr>
<td>130.0</td>
<td>133.57</td>
</tr>
</tbody>
</table>

Where: X = TIME (min)    Y = BODY WEIGHT (lbs)

Would you like to plot some other data?
If YES, please enter Y; if NO, please enter N:  > Y

1) Body Temperature vs. Time
2) Heat Loss vs. Time
3) Body Weight vs. Time
4) Skin Temperature vs. Time

Please enter 1, 2, 3, or 4:  > 4
The output file will be named SKINTEMP.TXT

<table>
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<th>Y</th>
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<td>102.44</td>
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<tr>
<td>125.0</td>
<td>102.55</td>
</tr>
<tr>
<td>130.0</td>
<td>102.65</td>
</tr>
</tbody>
</table>

Where: \( X = \text{T} \text{IME (min)} \quad Y = \text{SKIN TEMP. (Deg. F)} \)

Would you like to plot some other data?
If YES, please enter Y; if NO, please enter N:  > n

Would you like to test some other conditions?
If YES, please enter Y; if NO, please enter N:  > n
APPENDIX 4
Chatterjee, Sharmista
Seagrave, Richard C.

Presented at the 1993 International Conference on Environmental Science—July 1993

An estimate of the second law thermodynamic efficiency of the various units comprising an environmental control + life support system (ECLS)
ABSTRACT

The objective of this paper is to present an estimate of the second law thermodynamic efficiency of the various units comprising an Environmental Control and Life Support Systems (ECLSS). The technique adopted here is based on an evaluation of the 'lost work' within each functional unit of the subsystem. Pertinent information for our analysis is obtained from a user interactive integrated model of an ECLSS. The model was developed using ASPEN. A potential benefit of this analysis is the identification of subsystems with high entropy generation as the most likely candidates for engineering improvements.

THIS WORK HAS been motivated by the fact that the design objective for a long term mission should be the evaluation of existing ECLSS technologies not only the basis of the quantity of work needed for or obtained from each subsystem but also on the quality of work.

In a previous study Brandhorst [1] showed that the power consumption for a partially closed and a completely closed regenerable life support systems were estimated as 3.5 kw/individual and 10-12 kw/individual respectively. With the increasing cost and scarcity of energy resources, our attention is drawn to evaluate the existing ECLSS technologies on the basis of their energy efficiency. In general the first law efficiency of a system is usually greater than 50%, [2]. From literature, the second law efficiency is usually about 10%, [3]. The estimation of second law efficiency of the system indicates the percentage of energy degraded as irreversibilities within the process. This estimate offers more room for improvement in the design of equipment.

From another perspective, our objective is to keep the total entropy production of a life support system as low as possible and still ensure a positive entropy gradient between the system and the surroundings. The reason for doing so is as the entropy production of the system increases, the entropy gradient between the system and the surrounding decreases, and the system will gradually approach equilibrium with the surroundings until it reaches the point where the entropy gradient is zero. At this point no work can be extracted from the system. This is called as the ' dead state ' of the system, [4].

METHODS OF SECOND LAW ANALYSES :

The irreversibilities or entropy generation within a process is evaluated on the basis of the second law of thermodynamics using two widely used techniques. These are:
- Availability Analysis/Exergy Analysis
- Lost Work Analysis

Availability analysis or exergy analysis is a widely used technique. This was first proposed by Guoy and Stodola [5,6]. Availability is a measure of the useful work potential of a stream which is at a different state other than the environment. This concept has been extensively used in determining the efficiency in areas ranging from space heating to cryogenic processes, [7,8].

The method of Lost Work Analysis was first proposed by Seader [9].

For our study we chose the lost work approach as it provides a more intuitive feeling for the irreversibilities within a functional unit.

CONCEPT OF LOST WORK ANALYSIS :

The basic requirement of the second law is :

The total entropy change of an isolated system,

\[ \Delta S_{\text{sys}} \geq 0.0 \] (1)

For a control volume with a steady state process where the surroundings are at a temperature of T_0, the rate of change of the total entropy of the system is given by, [10]
The rate of change of the total entropy of the system = Net rate of entropy transfer by flowing streams + Rate of entropy exchange with the surroundings from heat transfer

Thus

\[ \Delta(Sm)_{fs} - \frac{Q}{T_0} > 0.0 \]  

where \( \Delta(Sm)_{fs} \) is the difference in entropy between the feed and the product.

\( Q \) is the net heat transfer from the system to the surroundings.

For a steady state flow process there is no internal energy accumulation within the system. The law of conservation of energy can be expressed as,

\[ \Delta \left( H + \frac{1}{2} u^2 + zg \right)_{fs} = Q - W_S \]  

where \( \Delta \left( H + \frac{1}{2} u^2 + zg \right)_{fs} \) is the difference in energy between the inflow and the outflow streams.

\( Q \) is the heat flow into the system.

\( W_S \) is the work done by the system.

For any system which requires work the amount of work required will be a minimum if the system undergoes a reversible change. This minimum work required is called the "ideal work", \( (W_{ideal}) \). Since there is no degradation of work the entropy generation for a reversible process is equal to zero and equation (2) becomes

\[ Q = T_0 \Delta(Sm)_{fs} \]

Substituting the above value in equation (3) and rearranging gives

\[ W_{ideal} = T_0 \Delta(Sm)_{fs} - \Delta \left( H + \frac{1}{2} u^2 + zg \right)_{fs} \]  

(4)

In most processes the kinetic and potential energy terms are negligible, and equation (4) can be written as

\[ W_{ideal} = T_0 \Delta(Sm)_{fs} - (\Delta H)_{fs} \]  

(5)

It is justified to mention here that from an availability viewpoint, according to the definition of \( W_{ideal} \) as given in (5), the minimum work required is equivalent to the difference in availability between the input and the output streams.

For a work producing process the "lost work" is the work which is lost due to irreversibilities within the process. It is expressed as the difference between the ideal work which could be produced by the process \( (W_{ideal}) \) and the actual work produced by the process \( (W_S) \). Thus from equations (3) and (5), \( W_{lost} \) can be written as

\[ W_{lost} = T_0 \Delta(Sm)_{fs} - Q \]  

(6)

Conventionally, there are two kinds of processes. A spontaneous process is one which produces work, i.e \( W_{ideal} \) is positive. Then

\[ W_S = W_{ideal} - W_{lost} \]  

(7)

A nonspontaneous process is one which requires some form of external work to be supplied, i.e \( W_{ideal} \) is negative. Hence

\[ |W_S| = |W_{ideal}| + W_{lost} \]  

(8)

Thereby the second law efficiency for each type of process can be defined as

\[ \eta_2 (\text{spontaneous process}) = \frac{W_S}{W_{ideal}} \]  

(9)

\[ \eta_2 (\text{nonspontaneous process}) = \frac{W_{ideal}}{W_S} \]  

(10)

**METHODOLOGY ADOPTED:**

The following steps were performed to evaluate the lost work and second law efficiency of each subsystem of an ECLSS based on the above developed concepts:

- Defining the subsystem boundary of the ECLSS subsystem chosen, i.e identifying the input and output streams from the subsystem chosen.

- Choosing the reference temperature. For our study we chose the space craft cabin temperature of 70°F.

- Evaluating the \( W_{lost} \) within each functional unit of the ECLSS subsystem.

- Depending on the subsystem information provided, we calculated the \( W_{ideal} \) for each subsystem.

- Knowing \( W_{lost} \) and \( W_{ideal} \), the second law efficiency is thus evaluated.

It is to be noted that in order to calculate \( W_{lost} \) and \( W_{ideal} \), detailed information about the input and output streams is required, i.e enthalpy, entropy and mass flow rate. To obtain the necessary data we developed a pseudo steady state model of the ECLSS using a state of the art chemical process simulator called ASPEN (Advanced System for Process Engineering). The underlying feature of our model is...
that the inputs to the ECLSS subsystems are regulated by a user interactive model of the crew in the space craft. The crew model is appended to the main ASPEN code. The capability of the crew model is that given crew specifications of age, weight, gender and activity level, the model can be used to compute the flow rates of the different waste streams from the crew, which then serve as inputs to the ECLSS.

RESULTS:

A conventional ECLSS consists of the following subunits [11]:
- Solid waste management
- Humidity condensate removal
- Trace removal subsystem
- CO2 reduction subsystem
- CO2 removal subsystem
- O2 generation subsystem
- Water recovery subsystem

Presented herein are the results obtained for a few sample technologies which are commonly in use in an ECLSS design.

WET OXIDATION OF SOLID WASTE:

The solid waste is oxidized in an autoclave at a pressure of 1067 psia in the presence of water, [12]. The reaction temperature depends on the carbon content of the feed and is usually between 100 and 374 °C. For our modeling purposes we assume that only the carbon content of the feed is oxidized.

For the ASPEN model the reactor is modeled using a RSTOIC block as shown in Figure 1. The reactor temperature is set according to the carbon content of the feed using the linear relationship as given by Takahashi [12]

% of carbon in feed = -0.65 × oxidation + 194 temperature

The mass and energy balance information of the streams involved in the process is shown in Table 1. It is to be noted that all these values are for a basal case of a crew consisting of 1 man of age 25 years weighing 60 kg resting at basal metabolic rate.

<table>
<thead>
<tr>
<th>Stream</th>
<th>Mass flow rate (lb/hr)</th>
<th>Total enthalpy (Btu/hr)</th>
<th>Total entropy (Btu/hr•R)</th>
</tr>
</thead>
<tbody>
<tr>
<td>S1</td>
<td>3.44×10^-4</td>
<td>12.91×10^-4</td>
<td>23.56×10^-7</td>
</tr>
<tr>
<td>S2</td>
<td>9.16×10^-4</td>
<td>-1.4×10^-3</td>
<td>-2.64×10^-3</td>
</tr>
<tr>
<td>S3</td>
<td>1.26×10^-3</td>
<td>-4.77</td>
<td>-3.63×10^-6</td>
</tr>
<tr>
<td>S4</td>
<td>1.26×10^-3</td>
<td>-4.85</td>
<td>1.72×10^-5</td>
</tr>
<tr>
<td>S5</td>
<td>0</td>
<td>0</td>
<td>0</td>
</tr>
<tr>
<td>Q1</td>
<td></td>
<td>Heat duty = -4.77 Btu/hr</td>
<td></td>
</tr>
<tr>
<td>Q2</td>
<td></td>
<td>Heat duty = -0.08 Btu/hr</td>
<td></td>
</tr>
</tbody>
</table>

The lost work analysis of each unit operation block comprising the Solid waste oxidizer is given in Table 2.

<table>
<thead>
<tr>
<th>Unit operation block</th>
<th>Lost Work (Btu/hr)</th>
<th>% of total lost work</th>
</tr>
</thead>
<tbody>
<tr>
<td>Wet oxidizer</td>
<td>4.71</td>
<td>98.1</td>
</tr>
<tr>
<td>Depressurizer</td>
<td>0.091</td>
<td>1.9</td>
</tr>
<tr>
<td>Total lost work</td>
<td>4.801</td>
<td></td>
</tr>
</tbody>
</table>

\[ Wideal = 530 \left(1.72\times10^{-5} + 26.36\times10^{-7} - 23.56\times10^{-7} \right) \times \left(-4.85 - 12.91\times10^{-4} + 14.05\times10^{-4}\right) \]
\[ = 4.86 \text{ Btu/hr} \]

Since \( Wideal \) being positive, this is a spontaneous process. Hence
\[ \eta_2 = \frac{0.059}{4.86} \]
\[ = 1.21\% \]
The Catalytic oxidation process is chosen for trace contaminant removal. This design was proposed by Ammann, [13]. In this subsystem, the incoming trace gas is split into two fractions depending on its methane content (i.e. higher the methane content, the greater the volume of gas which goes into the high temperature oxidizer). One portion of the incoming trace gas is oxidized in a high temperature catalytic oxidizer (HTCO) which is maintained at a temperature of 400 - 450 °C. Prior to entering the HTCO the gas is heated in an electric heater. Remaining portion of the gas is oxidized in a low temperature catalytic oxidizer (LTCO) which is maintained at ambient temperature of 70 °F. ASPEN model of this subsystem is shown in Figure 2.

ASPEN results for this model is shown in Table 3.

Table 3: Stream properties of Catalytic oxidation subsystem

<table>
<thead>
<tr>
<th>Stream</th>
<th>Mass flow rate (lb/hr)</th>
<th>Total enthalpy (Btu/hr)</th>
<th>Total entropy (Btu/hr °R)</th>
</tr>
</thead>
<tbody>
<tr>
<td>S1</td>
<td>35.66</td>
<td>-3291.38</td>
<td>1.17</td>
</tr>
<tr>
<td>S2</td>
<td>3.45</td>
<td>-318.44</td>
<td>0.113</td>
</tr>
<tr>
<td>S3</td>
<td>3.45</td>
<td>135.9</td>
<td>0.71</td>
</tr>
<tr>
<td>S4</td>
<td>3.45</td>
<td>309.64</td>
<td>0.86</td>
</tr>
<tr>
<td>S5</td>
<td>3.45</td>
<td>309.33</td>
<td>0.86</td>
</tr>
<tr>
<td>S6</td>
<td>3.45</td>
<td>-145.0</td>
<td>0.39</td>
</tr>
<tr>
<td>S7</td>
<td>32.21</td>
<td>-2972.98</td>
<td>1.05</td>
</tr>
<tr>
<td>S8</td>
<td>32.21</td>
<td>-2975.56</td>
<td>1.05</td>
</tr>
<tr>
<td>S9</td>
<td>35.66</td>
<td>-3120.25</td>
<td>1.49</td>
</tr>
</tbody>
</table>

Q1 Heat duty: 173.7 (Btu/hr)
Q2 Heat duty: -0.297 (Btu/hr)
Q3 Heat duty: -2.56 (Btu/hr)

The lost work analysis of each unit operation block comprising the Catalytic Oxidizer is shown in Table 4.

<table>
<thead>
<tr>
<th>Unit operation block</th>
<th>Lost Work (Btu/hr)</th>
<th>% of total lost work</th>
</tr>
</thead>
<tbody>
<tr>
<td>Diverter</td>
<td>0</td>
<td>0</td>
</tr>
<tr>
<td>Heat Exchanger</td>
<td>67.31</td>
<td>38.2</td>
</tr>
<tr>
<td>Heater</td>
<td>79.5</td>
<td>45.1</td>
</tr>
<tr>
<td>HTCO</td>
<td>0.297</td>
<td>0.17</td>
</tr>
<tr>
<td>LTCO</td>
<td>2.56</td>
<td>1.45</td>
</tr>
<tr>
<td>Mixer</td>
<td>26.5</td>
<td>15.04</td>
</tr>
<tr>
<td>Total lost work</td>
<td>176.17</td>
<td></td>
</tr>
</tbody>
</table>

\[ W_{\text{ideal}} = 530 \left( 1.49 - 1.17 \right) - \left( -3120.25 + 3291.38 \right) \]
\[ = -1.53 \text{ Btu/hr} \]

This is a nonspontaneous process. Hence
\[ W_S = 1.53 + 176.17 \]
\[ = 177.7 \text{ Btu/hr} \]
\[ \eta = \frac{1.53}{177.7} = 0.86\% \]

BOSCH SUBSYSTEM:
The Bosch subsystem was chosen for CO₂ reduction. The design for our model is based on a design proposed by Minemoto et al. [14]. A series of two reactors are used. The first reactor is at a temperature of 1300 [K] while the other reactor is operating at a temperature of 900 [K]. The reactions are given by:

Reactor 1
CO₂ + H₂ ⇌ CO + H₂O (12)

Reactor 2
CO + H₂ ⇌ C + H₂O (13)

Since all the CO₂ is eventually converted to C a stoichiometric ratio of H₂ to CO₂ is maintained.

The properties of the streams present in the Bosch subsystem as obtained from our model is shown in Table 5.

Table 5: Stream properties of Bosch subsystem

<table>
<thead>
<tr>
<th>Stream</th>
<th>mass flow rate (lb/hr)</th>
<th>total enthalpy (Btu/hr)</th>
<th>total entropy (Btu/hr °R)</th>
</tr>
</thead>
<tbody>
<tr>
<td>S1</td>
<td>0.048</td>
<td>-183.79</td>
<td>-6.04 × 10⁻⁴</td>
</tr>
<tr>
<td>S2</td>
<td>4.37 × 10⁻³</td>
<td>-54.16 × 10⁻⁶</td>
<td>-5.98 × 10⁻⁷</td>
</tr>
<tr>
<td>S3</td>
<td>0.398</td>
<td>-677.1</td>
<td>-0.139</td>
</tr>
<tr>
<td>S4</td>
<td>0.398</td>
<td>515.04</td>
<td>0.778</td>
</tr>
<tr>
<td>S5 solid</td>
<td>1.3 × 10⁻²</td>
<td>4.51</td>
<td>4.27 × 10⁻³</td>
</tr>
<tr>
<td>S5 vapor</td>
<td>0.385</td>
<td>-130.4</td>
<td>0.436</td>
</tr>
<tr>
<td>S6</td>
<td>3.9 × 10⁻²</td>
<td>-268.11</td>
<td>-0.087</td>
</tr>
<tr>
<td>S7 solid</td>
<td>1.3 × 10⁻²</td>
<td>-0.08</td>
<td>-1.5 × 10⁻⁴</td>
</tr>
<tr>
<td>S7 vapor</td>
<td>0.346</td>
<td>-507.11</td>
<td>-0.177</td>
</tr>
<tr>
<td>S8</td>
<td>1.3 × 10⁻²</td>
<td>-0.015</td>
<td>-2.87 × 10⁻⁵</td>
</tr>
<tr>
<td>S9</td>
<td>0.398</td>
<td>-677.1</td>
<td>-0.139</td>
</tr>
<tr>
<td>Q1</td>
<td>Heat duty: 1192.14 Btu/hr</td>
<td></td>
<td></td>
</tr>
<tr>
<td>Q2</td>
<td>Heat duty: -640.93 Btu/hr</td>
<td></td>
<td></td>
</tr>
<tr>
<td>Q3</td>
<td>Heat duty: -649.41 Btu/hr</td>
<td></td>
<td></td>
</tr>
</tbody>
</table>

Table 6: Lost work analysis of unit operation blocks of Bosch subsystem

<table>
<thead>
<tr>
<th>Unit operation block</th>
<th>Lost work (Btu/hr)</th>
<th>%lost work</th>
</tr>
</thead>
<tbody>
<tr>
<td>Mixer</td>
<td>6.89</td>
<td>0.55</td>
</tr>
<tr>
<td>Reactor 1</td>
<td>486.01</td>
<td>39.04</td>
</tr>
<tr>
<td>Reactor 2</td>
<td>461.95</td>
<td>37.11</td>
</tr>
<tr>
<td>Condensate Remover</td>
<td>276.13</td>
<td>22.18</td>
</tr>
<tr>
<td>Solid carbon remover</td>
<td></td>
<td></td>
</tr>
<tr>
<td>Total lost work</td>
<td>1244.76</td>
<td></td>
</tr>
</tbody>
</table>

\[ W_{\text{ideal}} = 530 \cdot (2.87 \times 10^{-5} - 0.087 + 6.21 \times 10^{-4} + 5.98 \times 10^{-7} - 1.72 \times 10^{-5}) - (286.11 - 0.015 + 178.94 + 54.16 \times 10^{-6} + 4.85) \]
\[ = 38.53 \text{ Btu/hr} \]

Hence this is a spontaneous process.

\[ W_s = 38.53 - 1244.76 \]
\[ = -1206.23 \]

This is an example of a highly nonideal process. The actual process is spontaneous however under simulated working conditions the process requires external work to be supplied.

STATIC FEED WATER ELECTROLYSIS:

This technique is used for O₂ generation. Our model is based on a design proposed by Fortunato et al [15]. In this method the water from the water
The regeneration system is electrolyzed in a water retention matrix to produce \( \text{H}_2 \) and \( \text{O}_2 \). The electrolysis chamber is maintained at 30 psia. The ASPEN model for the process is shown in Figure 4. The water from the water regeneration unit is pressurized to 30 psia before being fed into the electrolysis chamber. A cooler is used prior to the electrolysis chamber to remove the heat generated due to compression of water. The compressed water is cooled to ambient temperature before being fed into the electrolysis cell. Since there are no tailor made blocks to simulate the reactions occurring at the electrodes in ASPEN, the reaction was simulated using a RSTOIC block.

<table>
<thead>
<tr>
<th>Unit operation</th>
<th>Lost work (Btu/hr)</th>
<th>% lost work</th>
</tr>
</thead>
<tbody>
<tr>
<td>Pump</td>
<td>0.427</td>
<td>0.38</td>
</tr>
<tr>
<td>Cooler</td>
<td>0.03</td>
<td>0.03</td>
</tr>
<tr>
<td>Electrolyzer</td>
<td>96.99</td>
<td>87.08</td>
</tr>
<tr>
<td>Gas separator</td>
<td>13.93</td>
<td>12.51</td>
</tr>
<tr>
<td><strong>Total lost work</strong></td>
<td><strong>113.38</strong></td>
<td></td>
</tr>
</tbody>
</table>

Table 8 presents an analysis of the lost work within each subsystem.

Thus this is an example of a nonspontaneous process.

\[
W_{\text{ideal}} = 530 \left( -4.76 \times 10^{-4} - 2.46 \times 10^{-4} + 0.21 \right) - (-0.13 - 0.25 + 654.8) = -543.5 \text{ Btu/hr}
\]

**Humidity Condensate Separator:**

In order to maintain the cabin humidity at a desired level, the exit air from the trace removal subsystem is cooled to a temperature such that the amount of moisture in the air leaving the condenser separator is equal to the amount of moisture desired in the cabin. The condensate is removed and treated to obtain water of potable quality. For the ASPEN model the condenser separator is modeled as a single unit using a SEP block. The temperature of the condenser is set in a user incorporated Fortran subroutine which takes into account the humidity levels of the incoming and outgoing streams.

Table 7: Properties of streams present in the Water Electrolysis subsystem

<table>
<thead>
<tr>
<th>Stream</th>
<th>Mass flow rate (lb/hr)</th>
<th>Total enthalpy (Btu/hr)</th>
<th>Total entropy (Btu/hr °R)</th>
</tr>
</thead>
<tbody>
<tr>
<td>S1</td>
<td>9.6\times10^{-2}</td>
<td>-654.8</td>
<td>-0.21</td>
</tr>
<tr>
<td>S2</td>
<td>9.6\times10^{-2}</td>
<td>-654.83</td>
<td>-0.21</td>
</tr>
<tr>
<td>S3</td>
<td>9.6\times10^{-2}</td>
<td>-654.86</td>
<td>-0.21</td>
</tr>
<tr>
<td>S4</td>
<td>9.6\times10^{-2}</td>
<td>-0.38</td>
<td>-0.27</td>
</tr>
<tr>
<td>S5</td>
<td>1.07\times10^{-2}</td>
<td>-0.25</td>
<td>-4.76\times10^{-4}</td>
</tr>
<tr>
<td>S6</td>
<td>8.52\times10^{-2}</td>
<td>-0.13</td>
<td>-2.46\times10^{-4}</td>
</tr>
</tbody>
</table>

Q1 Heat duty: -0.03 Btu/hr
Q2 Heat duty: 654.55 Btu/hr
Table 9: Stream properties of streams occurring in Humidity Condense Separator

<table>
<thead>
<tr>
<th>Stream</th>
<th>Mass flow rate (lb/hr)</th>
<th>Total enthalpy (Btu/hr)</th>
<th>Total entropy (Btu/hr °R)</th>
</tr>
</thead>
<tbody>
<tr>
<td>S1</td>
<td>35.66</td>
<td>-3120.22</td>
<td>1.49</td>
</tr>
<tr>
<td>S2</td>
<td>7.18 x 10^-2</td>
<td>-489.58</td>
<td>-0.154</td>
</tr>
<tr>
<td>S3</td>
<td>35.59</td>
<td>-2783.35</td>
<td>1.35</td>
</tr>
<tr>
<td>Q1</td>
<td>Heat duty:</td>
<td>-152.71</td>
<td></td>
</tr>
</tbody>
</table>

Since the heat of condensation is not being used to do any useful work, hence

Lost work = 530 (1.35 +0.154 -1.49) + 152.71
= 160.13 Btu/hr

Wideal = 530(1.35 +0.154 - 1.49) - (-489.58 - 2783.35 +3120.22)
= 160.13 Btu/hr

This is a spontaneous process.

Ws = 0

Thus n2 = 0

This is valid since we had assumed that the heat of condensation is not used for doing useful work, thereby the amount of work which can be obtained from the process is zero.

CONCLUSIONS:

As shown in Tables 2, 4 and 6 lost work analysis provides a means of identifying areas within a subsystem where maximum work is lost due to irreversibilities. The magnitude of second law efficiency of some subsystems like the Catalytic Oxidation process or the Bosch subsystem reveal that these subsystems merit considerable attention to redesign their functional units from an energy conservation viewpoint.

For example, our study shows that percentage of lost work in the reactors used for CO2 reduction is between 35 to 39 %. It is hopeful that this will provide

The overall conclusion to be drawn from this paper is that there exists a potential for energy conservation in the currently used ECLSS technologies which warrants evaluation of the present operating conditions.

REFERENCES:


Figure 1: Schematic of Solid Waste Wet Oxidation Subsystem.

Figure 2: Schematic of Catalytic Oxidation Process
Figure 3: Schematic of Bosch subsystem

Figure 4: Schematic of Water Electrolysis process
Figure 5: Schematic of Humidity Condensate Separator