Anticipated Mission Tasks

The original FTS concept for Space Station Freedom (SSF) was to provide telerobotic assistance to enhance crew activity and safety and to reduce crew EVA (Extra Vehicular Activity) activity. The first flight of the FTS manipulator systems would demonstrate several candidate tasks and would verify manipulator performance parameters. These first flight tasks included unlocking a SSF Truss Joint, mating/demating a fluid coupling, contact following of a contour board, demonstrating peg-in-hole assembly, and grasping and moving a mass. Future tasks foreseen for the FTS system included ORU (Orbit Replaceable Unit) change-out, Hubble Space Telescope Servicing, Gamma Ray Observatory refueling, and several in-situ SSF servicing and maintenance tasks. Operation of the FTS was planned to evolve from teleoperation to fully autonomous execution of many tasks.

This wide range of mission tasks combined with the desire to evolve toward full autonomy forced several requirements which may seem extremely demanding to the telerobotics community. The FTS requirements appear to have been created to accommodate the open-ended evolution plan such that operational evolution would not be impeded by function limitations. A recommendation arising from the FTS program to remedy the possible impacts from such ambitious requirements is to analyze candidate robotic tasks. Based on these task analyses, weigh operational impacts against development impacts prior to requirements definition. Many of the FTS requirements discussed in the following sections greatly influenced the development cost and schedule of the FTS manipulator. The FTS manipulator has been assembled at Martin Marietta and is currently in testing. Successful component tests indicate a manipulator which achieves unprecedented performance specifications.

Functional Requirements

The functional requirements of the manipulator involve environmental, performance, safety, and resource effects. Many of these requirements are driven by the space environment, such as operation in thermal extremes, the need for safety, and limited resource availability (weight and power). Most of these requirements, however, focus on the manipulator and component functions to ensure superior performance and ability to upgrade (evolution toward autonomy).

The primary robotic function of the FTS manipulator is that it move or manipulate objects in zero-gravity. Because interchangeable end-effectors were being considered, the manipulator requirements specify the tool-plate as the point of reference. The tool plate is the attachment point for the wrist force/torque sensor. A manipulated object's mass may be as high as 37 slugs (1200 lb.) with the manipulator able to move masses less that 2.8 slugs (90 lb.) at velocities of 6 inch/second. Unloaded tool plate velocity will be at least 24 inch/second. Accuracy of tool-plate positioning relative to the manipulator base frame must be within 1 inch and _+ 3 degrees. The manipulator must be able to resolve tool-plate incremental motion within 0.001 inch and 0.01 degrees. Additionally, repeatability must be within 0.005 inch and _+ 0.05 degrees with respect to the manipulator base frame. To perform useful work, the FTS manipulator was required to provide 20 pounds force and 20 foot-pounds torque output at the tool plate in any direction and in any manipulator configuration. These output force and positioning requirements were to be utilized with several control schemes including joint control, Cartesian control, and impedance control.
To operate in space, the FTS manipulator had to meet the shuttle safety requirements as well as the environmental extremes. The safety requirements, as discussed elsewhere in this paper, ensure Orbiter and crew safety through fault tolerance requirements. Safety is cited by Shattuck and Lowrie (1992) as "the single largest factor driving the system design." Safety and fault tolerance requirements resulted in monitoring of joint and Cartesian data, in checking of loop times to ensure proper functioning, in cross-strapping along communication paths, and in addition of a hardwire control capability as a backup operational mode. Orbiter launch and landing impart vibration into the system which requires structural analysis and testing. Electromagnetic interference (EMI) must be limited both from invading and from exiting the manipulator systems. However, the most demanding aspect of the space environment from the FTS designer's view is the thermal vacuum of space. Operation in a hard vacuum (10⁻⁵ torr) and over temperatures from -50°C to 95°C forces innovative designs, careful material selection, and extensive analysis.

Another consequence of the space environment is operation in zero-gravity. Designing the manipulator for a zero-g environment impacts structural, electromechanical, and electrical power considerations as well as the control system design. Because weight is a premium in space, motors are chosen to provide torques for zero-g operation. This saves significant weight and electrical power when compared to motors chosen for ground-based operation. Smaller motors also benefit the thermal control system. The structure must also be lightweight, which increases flexibility and lowers structural bending mode frequencies. While being lightweight and more flexible, space manipulators are expected to handle payloads more massive than the manipulator. This expectation is far different from terrestrial manipulators which usually handle payloads 1/10 their weight. To maintain stability and performance, a 10:1 ratio is maintained between the first bending mode and the control bandwidth. This ratio precludes use of high bandwidth PID servos used in more massive, terrestrial manipulators. To address the stability and performance issues in the FTS manipulator, the structure was designed for stiffness (12 Hz first bending mode) and the manipulator control has a 1.2 Hz bandwidth, an inertia decoupler, and joint-level torque, position, and velocity servo loops.

**Manipulator Design and Technologies**

Beyond safety, FTS manipulator design was driven by the thermal environment and the
positioning performance specifications. Of course, each manipulator subsystem was influenced by additional constraints and specifications. The following paragraphs describe the manipulator subsystem designs and technologies developed by Martin Marietta and its subcontractors to meet the FTS requirements. Manipulator subsystems discussed include manipulator kinematic design, link structure, actuators, control systems, and the end-of-arm tooling.

**Manipulator Kinematics**

A 7-DOF (degree-of-freedom) R-Y-P-P-Y-R design is used with the first joint (shoulder roll) utilized for task-dependent configuration optimization. The outer 6 joints are actively controlled for coordinated output motion. The kinematic design has few joint offsets and 90° twist angles to simplify the kinematics. The 6-DOF kinematic arrangement, with three adjacent pitch joints, provides a closed-form inverse kinematic solution with few singularities within the manipulator workspace. The singularities which occur when the wrist roll or wrist yaw align with the shoulder yaw are beyond the usual workspace of the manipulator. Other singularities occurring at joint limits and when the elbow passes over the "home" position, shown below, are eliminated with mechanical and software joint travel limits. The 3 inch displacement of the elbow joint is to allow the arm to fold back on itself for a greater workspace.

![FTS Manipulator - "Home" Position](image)

**Link Structure**

The manipulator links provide structural support as well as joint controller electronics packaging and thermal control. Packaging and thermal control determined link sizes while fracture and stiffness considerations drove the structural design of the links. A stiffness requirement of 1,000,000 pounds/foot and 1,000,000 foot-pounds/radian resulted in a smallest structural safety margin which exceeds 14, far greater than Shuttle requirement for a 1.4 factor of safety. Easy access to electronics is through side plates on the links. To avoid the cost and complication of active cooling, radiation is the primary thermal path. The controller boards sit in slots within the links which provide conduction paths to the link structure for radiation to the environment. The link designs use material coatings, mounting, and Kapton/Inconel film heaters to maintain thermal control.

**Actuators**

The joint actuator designs, developed by Martin Marietta and Schaeffer Magnetics, were also driven by positioning, performance, and thermal demands. These high-performance, zero backlash actuators each house a DC-motor, an harmonic drive transmission, an output torque sensor, an output position sensor, a fail-safe brake, hard-stops, and internally routed cabling. The design achieves considerable commonality between actuators. Three sizes are used - one for the 3 shoulder joints, one elbow joint, and one for the 3 wrist joints.

The DC-motors have brushless, delta-wound stators with samarium cobalt rotors. This design offers good thermal properties, low EMI, minimal rotational losses, and linear torque-speed relationships. Motor commutation signals are generated from Hall Effect sensors, a second set of which is installed for redundancy. A secondary set of windings within the stator, driven via an independent electrical path, provides at least 10% rated torque and 0.5 degrees/second joint velocity for operation of a backup mode. This degraded mode of operation, commanded joint-by-joint, satisfies the need for safing the manipulator after failure of a primary system. Fail-safe brakes attached to the motor rotor shaft are spring-loaded so that loss of power engages the brake. These brakes may be released with an EVA release bolt, which when turned 90° releases a cam
Harmonic drives provide 100:1 backdrivable gear reduction in a compact volume. The harmonic drives were chosen with HUIC-series cups and S-tooth profile teeth for torsional stiffness and zero backlash. Cup size is determined by joint torsional stiffness requirements. In fact, because of the relative flexibility of the harmonic drive, all other torsion members are considered rigid. Rather than the standard Oldham coupling to the wave generator, a specially designed cylindrical coupler was used to eliminate backlash. Additionally, the output is coupled to a flange around the motor and harmonic drive. This flange, mounted to large duplex bearings provides compactness, rigidity, and an efficient load path the output link.

An analog torque loop is implemented in the joint servos to accommodate the non-linear and high-frequency affects of the harmonic drives. Sensor values to the torque loop come from an output torque sensor embedded on the harmonic drive output flange. Strain gages are mounted to the spokes of the titanium flange. This sensor placement isolates the sensor from structural loads (bending), thus primarily transmitting actuator torque. For effective performance, this analog torque loop operates at 1500 Hz.

Like the manipulator structure, actuator housings and bearings were designed for stiffness and thermal stability. A standard bearing steel, 440C stainless, is used for all bearings. Bearing lubricant is Braycote 601, a liquid lubricant used in space applications. Its very low vapor pressure allows the actuator to not be sealed, but still designed to resist contamination and assembled in a clean room. The motor bearings are deep-groove roller bearings sized for the thrust load of brake engagement and spring pre-loaded to minimize temperature sensitivity. The output bearings are large diameter, duplex-pair, angular contact bearings (face-to-face mounting). These bearings share radial and thrust loads with another duplex-pair on the other side of the actuator. An exception is the wrist roll, which has a single, duplex pair mounted back-to-back for better rigidity against the bending moments of the full cantilever load. Unfortunately, this back-to-back installation has greater sensitivity to assembly misalignments. This sensitivity may contribute to the excessive, uncompensated friction discovered during recent wrist roll torque loop tests.

The actuator housings are aluminum and titanium. Titanium is utilized near bearings. The similar thermal properties of 440C stainless and 6Al-4V titanium minimize temperature effects on bearing pre-loads. These pre-loads were determined as a compromise between stiffness and friction drag. The actuator case was designed for thermal needs. Motor and brake heat is dissipated to the ends or to the casing and then radiated to the environment. Like the links, the actuator design uses thermal isolation, material coatings, and internally mounted film heaters to protect bearings from thermal gradients. These gradients could adversely affect actuator friction and positioning accuracy.

The positioning and incremental motion requirements call for encoder data within an arc-minute at resolutions to 22-bit sensor. To meet this need, inductive encoders were developed specifically for the FTS program by Aerospace Controls Corporation. These encoders have a fine and a coarse track used for incremental and absolute position resolution, respectively. Temperature effects on sensor accuracy were discovered during thermal testing. These errors were stable and repeatable with temperature, and are thus have been corrected in software.

All cabling in the manipulator is internally routed through links and actuators. Each actuator has a cable passageway designed to eliminate twisting of cabling and thus minimizing chafing opportunity. The innovative cabling within these actuators is of Flat Conductor Cables (FCC), manufactured by Tayco, Inc. FCC is used in space applications, but for this application up to 34 layers of laminated cables are used in a single actuator passageway. The cables consist of alternating layers of Kapton, FEP, and photo etched copper conductors with a vapor-deposited copper shield. These cables are to operate from -50°C to 95°C through thousands of cycles. These cables rout serial data, video signals, power, and discrete signals. Acceptance tests of a few cables indicated minor lamination problems apparently due to entrapped water vapor. Investigation of the cable manufacture and test indicated several areas for possible change as well as a method for cable repair. Recent cable tests to 100,000 mechanical cycles over full temperature ranges verified continued cable functionality.
Control Systems
The FTS manipulator control design provides 6-DOF active control over a wide range of payloads as well as impedance control for stable contact. The control algorithms are specified according to the NASREM architecture (NASA/NBS Standard Reference Model for Robotic Systems). NASREM is implemented as a layered architecture with 4 levels: Task, Elemental-Move, Primitive, and Servo. Use of these levels allows operation from teleoperation, the Servo level, advancing to fully autonomous task sequencing, the Task level. Developments to date have focused on the Servo level commands. The Servo level receives Cartesian manipulator commands and transforms them to joint level servos commands. Efforts with the NASREM Primitive level have incorporated point-to-point Cartesian path generation.

The wide payload range specified for the FTS manipulator causes the manipulator joints to experience inertial loads over several orders of magnitude. These loads are induced by the coupling which occurs between joints and affects the trajectory-tracking accuracy of the manipulator. The position controller implemented in the FTS manipulator compensates for these torques with a model-based inertial decoupler. The feed-forward decoupling scheme computes expected inertial torques due to commanded motion and sums this torque with the joint command. The position-dependent inertia matrices used to calculate these torques are computed every 200 ms. This value was chosen as a compromise of accuracy and computational burden.

In addition to the free-space performance requirements, satisfied with the position controller and inertial decoupler, the FTS manipulator must provide stable contact with its impedance control. The impedance controller is position-based, that is, the manipulator and joints are treated as actuators of Cartesian position. Thus, end-effector force measurements are transformed into Cartesian motion commands based on a desired output impedance. This approach was chosen over a torque-based approach because a torque-based approach has instabilities for higher stiffness values and may have difficulty applying large forces to a worksite. Also, a torque-based approach may store energy, resulting in large accelerations when contact is broken. To maintain stability during the transition from free-space motion to contact, a joint velocity feedback term is included for "augmented damping." The resulted lightly damped contact insures stability, but when contact is broken the free-space motion becomes overdamped and sluggish. A feed-forward velocity term is implemented to compensate for this poor free-space response. These control schemes, which increase the complexity of the controller are designed to meet the FTS free-space motion, payload capacity, and contact performance requirements.

An emergency shutdown (ESD) system is embedded in the manipulator control architecture. This system was implemented to provide active control of hazards to meet the payload safety requirement to be two-fault tolerant against catastrophic hazards. The primary hazards in this case are unplanned contact and excessive force generation. The ESD approach is to use 3 control levels to monitor joint and Cartesian positions and velocities, comparing both commands and sensor feedback. A separate ESD bus, which connects the joint, manipulator, and power controllers, is the path by which an ESD is initiated - removing power from the manipulator systems. The first level checks that commanded values are within allowable limits both in the manipulator controller and the joint controllers. The second level monitors safety critical parameters such as position, velocity, and torque with the joint controllers and within the manipulator controller collision avoidance routines. The final level of ESD monitoring is a check of redundant safety critical parameters in the redundant manipulator controller and in independent joint controllers.

In the event of an apparent failure, several possible ESD actions may be automatically initiated. The operator, of course, has a manual ESD to power off the manipulator at any time. If monitored values are elevated but do not pose immediate danger, a soft stop is initiated by the control software. A soft stop commands the manipulator to hold the current position with brakes off (disengaged). An example of a soft stop condition is a Cartesian manipulator command which violates a warning boundary near a known obstacle. A hardware ESD is initiated by any controller when an analog sensor value exceeds its limit - resulting in an ESD notification on the ESD bus. These analog comparisons are being performed at 1500 Hz. A software ESD occurs when a controller CPU detects an out-of-limit condition and signals the power module over the Mil-Std-1553B communication bus. The power module then initiates a combination ESD to power off the manipulator. A combination ESD is detected by software comparisons in the controllers and
initiates a software reset of a hardware limit value to force a hardware ESD. All these ESD paths were analyzed to determine reaction times to various failures such as a joint runaway. Hardware ESD's occur in 11 msec, combination ESD's occur in 30 to 206 msec, and a combination ESD may take up to 4026 msec for an over-temperature condition.

**Gripper/End-of-Arm Tooling**

The end-of-arm tooling built for the FTS manipulator has a parallel jaw gripper and space for later addition of an end-effector exchange mechanism. The gripper fingers are a cruciform designed for positive contact and retention because the gripper is backdrivable. The gripper fingers ride on a rack and pinion driven by a harmonic drive transmission and a single DC-motor. A pair of fail-safe brakes are installed to provide fault tolerance against inadvertent release. Brake failure or brake command failure results in a brake defaulting to its engaged position. Each of the two brakes can withstand forces greater than expected gripper forces (maximum anticipated load is 30 lb, brake hold is 50 lb.). Gripper forces are measure by a torque sensor and also by motor currents. The concern over inadvertent release also impacted the design of the planned task items. These items were instrumented to insure positive grasp. As a final safety measure, the gripper fingers are attached with EVA compatible bolts which may be removed on-orbit to release the gripper.

**SAFETY REQUIREMENTS**

Robotic Manipulator Systems can provide the capability to perform work and assist humans in space as long as they are safe and reliable. The space based requirements differ significantly from terrestrial based manipulators used in industry and research. In most terrestrial robot implementations, the prime method for dealing with failures is to keep workers out of the robot workspace when active and by accepting the occasional parts damage following a failure due to high volume parts fabrication. This approach is not acceptable for space applications where humans are involved, and the effect impacts the design requirements for space manipulator systems.

**Hazards and Controls**

All manned space flight systems are assessed for flight hazards their use imposes. From such an assessment the causes of those hazards are determined, and methods to control those hazards are developed. To gain flight acceptance, multiple levels of hazard control must be designed for and verified for assuring the desired level and coverage of controls. In the FTS system development, safe control of hazardous operations forced additional requirements in the design of the manipulator system, its interfaces with the Orbiter and the task elements the FTS was to demonstrate interaction with.

The primary hazards associated with the FTS manipulator operations and the three methods for providing safe control are listed:

A) Unplanned contact or impact during operations
   1) Operator and computer control to not command unplanned contact.
   2) Boundary management software operation.
   3) Redundant boundary management software operation in the safety computer

B) Inadvertent release of hardware
   1) Hardwired enable gripper brake power from independent switch in the aft flight deck
   2) PGSC (Portable General Support Computer: laptop computer) command to release gripper Brake #1
   3) Hand controller switch to release gripper Brake #2

C) Failure to stow for safe Orbiter landing
   1) Normal computer operations (With hardwired control for added reliability)
   2) Jettison via RMS (or EVA if time permits)
   3) EVA operations to stow or jettison

D) Excessive applied gripper force or torque
   1) Force control using gripper force sensor
   2) Current limiting ESD (Emergency shutdown detection)
   3) Redundant current limiting ESD

E) Excessive applied manipulator force or torque
   1) Normal control with active Cartesian load from joint torque command
   2) Cartesian force limiting, using wrist force/torque sensor channel A
   3) Redundant Cartesian force limiting, using wrist force/torque sensor channel B.
**Mission Operation To Control Hazards**

Primary concerns in the design of space manipulator systems have to do with the effects of system failures on the crew or vehicle. Operational limitations of use are placed on robotic systems that may otherwise be perfectly capable of performing their intended operations. Limitation on use are due to the fact that if a system is performing a task and were to have a failure, the effect of that failure must not prohibit the intended function from being performed in the time frame that that function is critically needed, and any failure must not prohibit any other safety related operations from being carried out during its time of criticality.

For a system to continue operations after a failure, any remaining operability the system might contain must also provide that same capability to make itself safe to the vehicle and crew if it were to suffer a failure. Otherwise that additional level of operability would only be allowed for temporary use to make the task situation safe, remove the robot from the task area, and then stow it in a safe returnable state or eject it so the vehicle can return to Earth. The added operability would not be allowed for continued use to proceed with the intended task, except to make the situation safe. This is the fundamental concept of hazard control for the Orbiter.

**FTS Fail Safe Operations**

Several FTS configuration descriptions follow below along with design features to address key functions which allow for safe operations. The designs comply with NASA’s Orbiter safety policy and requirements of NSTS 1700.7B with interpreted in NSTS 18798A. In several cases, the hardware or software system could not be designed to meet the required levels of fault tolerance without significantly complicating the design or dexterity of the manipulator system. Therefore reductions in compliance with the safety requirements placed operational limitations on the use of the FTS System. The system is considered fail safe; where under any failure the system will not cause a catastrophic hazard, and therefore does not jeopardize the safety of the Orbiter or crew. The FTS system is not fail-operational. Such a system, after any initial failure, could continue normal intended operations since it would still retain the ability to make itself safe after a second failure.

The DTF-1 concept fulfills the first method of hazard control for Orbiter safety using its normal modes of operation. If any of the single points of failure occur, normal operations will cease and an attempt to safe the manipulator system by use of the hardwired control. Note that hardwired control is only a supplement to the first level of hazard control. If the manipulator system cannot be safed by use of the hardwire control, the mission will be assessed to determine if enough time remains to perform an EVA to safe the manipulator system. If hardwired control cannot safe the manipulator system and time does not permit an EVA to safe the manipulator or remove it for stowage, then the RMS will grapple the telerobot using the RMS grapple fixture for jettison. This is the second method for hazard control. The third method of hazard control to provide two fault tolerance for Orbiter safety is EVA operations. Remedial operations could be to remove the manipulator, release the gripper and/or release the actuator brakes. This is to allow stowage of the manipulator, either into its caging devices or by removal and strapping it in the airlock, or otherwise by release into orbit.

**Hardwired Control**

The FTS system incorporates a backup hardwired control capability in the event of a failure which precludes closed loop computer control of the manipulator system. The main purpose is to minimize the likelihood of having to jettison the system or perform an EVA operation. This has the effect of making the computer system, sensor systems, software, servo systems and most other hardware single fault tolerant, even though the operations would be significantly degraded in performance.

Operational use of the hardwired control is limited to safing of the system after a failure, by stowing the arm to allow a safe Orbiter return. It allows operator control of individual manipulator joints for stowage and for gripper actuation in the event of computer control or motor drive failure. When selected, primary power is removed from all manipulator motor and brake drivers while retaining power to camera controls. Software recognizes the status of the hardwire control, and commands off all motors and brakes, so that return to normal computer operations after hardwired control starts with all motors and brakes powered off.
Hardwire control is limited to very low joint rates and torques in a two fault tolerant manner. Hardwired control is by sequential, joint-by-joint movement, and provided no force accommodation to minimize forces imparted into interfaces. Only a limited set of initiated tasks are likely to be able to be completed. Emergency shutdown detection (ESD) is not operational during hardwired control operation, as the operator can de-power the hardwire drive to stop payload motion, and brakes can also be used to stop motion.

**EVA Operations**
Several failures of components employ EVA as the third fault tolerant paths to ensure stowage of DTF-1 for safe return of the Orbiter. The manipulator actuators, gripper mechanism, and manipulator caging mechanisms represent major groups of such components.

Failure of a caging mechanism to release the arm for operation would not require EVA for safing the manipulator. EVA would be used as the third path for safing the manipulator if more than one of the four caging mechanism fail to close. In this case, removal of the manipulator at its shoulder interface and either manual release into orbit or stowage in the airlock would be required.

Failure of a manipulator actuator motor drive electrically or mechanically would require EVA as the third fault tolerant path. Mechanical release of the joint actuator brake allows EVA backdrive of the joint into the caging position. If a manipulator joint seizes, then EVA is employed as the third fault tolerant path to remove the manipulator at the shoulder and release into orbit or stowage in the airlock.

**Single-Points Failures**
There are several single point failures that remain in the FTS system which may lead to failure of the manipulator to complete a task, or to stow itself for a safe Orbiter return. For the Orbiter this is considered a catastrophic hazard, therefore the requirements for payloads to provide two fault tolerant methods of dealing with these effects.

The FTS single-point failures which lead to an EVA or jettison are few in function, but have commonality within the actuator and gripper. These failures are seized bearings or gears, a short within the motor winding, or a short or open in a brake winding.

**Safety Critical Subsystems**
The DTF-1 Flight Experiment of FTS has fifteen different safety critical subsystems and equipment groups, as listed.

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This is only a listing, descriptions of these subsystems will be presented in a future paper.
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