Behavior of Rapidly Sheared Bubble Suspensions

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ABSTRACT
An experiment to be carried out aboard the International Space Station is described. A suspension consisting of millimeter-sized bubbles in water containing some dissolved salt, which prevents bubbles from coalescing, will be sheared in a Couette cylindrical cell. Rotation of the outer cylinder will produce centrifugal force which will tend to accumulate the bubbles near the inner wall. The shearing will enhance collisions among bubbles creating thereby bubble phase pressure that will resist the tendency of the bubbles to accumulate near the inner wall. The bubble volume fraction and velocity profiles will be measured and compared with the theoretical predictions. Ground-based research on measurement of bubble phase properties and flow in vertical channel are described.

INTRODUCTION
Hydrodynamic interactions and direct particle collisions can have a dramatic influence on the flow properties of suspensions in which the disperse phase volume fraction is 0.1 or greater. Over the last thirty years analytical theories, numerical simulations, and careful experiments using well defined suspensions in simple flows have provided considerable progress toward a quantitative description of the rheology of concentrated, low Reynolds number (based on particle size and characteristic velocity) suspensions (i.e., Nott and Brady 1994). The effects of dispersed particles, drops, or bubbles with the high Reynolds number flow of a suspension is likely to be even more dramatic because of the enhanced transport of momentum due to Reynolds stresses. However, most previous theories of high Reynolds number suspensions have considered only interactions of particles with the mean flow while neglecting particle-particle interactions.

The treatment of particle interactions in inertial suspensions is more difficult than in viscous suspensions for several reasons. The equations of motion for a low Reynolds number fluid are quasisteady and linear, indicating that the flow responds instantaneously to changes in the particle configuration. This feature is essential to the methods of theoretical analysis and numerical simulations used to treat most low Reynolds number suspensions. A general, moderate or high Reynolds number flow is neither quasi-steady nor linear with the result that the fluid flow at any instant in time in inertial suspensions depends on the time history of the particle positions and velocities and one cannot use the analytical and numerical techniques developed for small Reynolds number, linearized equations of motion. As a result, little is known about the equations of motion for inertial suspensions of particles or drops.

However, there is a special case of an inertial suspension that is particularly amenable to theoretical analysis: a suspension of surfactant-free, spherical, high Reynolds number bubbles. In this case, the vorticity produced by the bubble motion is small and the flow induced by the bubbles may be described using the potential flow approximation. The fluid velocity may then be expressed as the gradient of potential obtained by solving Laplace’s equation. It is possible then to derive from first principles the equations of motion for this type of bubble suspension.
In addition, numerical simulation of motion of many interacting bubbles can be conducted to aid in developing the equations of motion for such bubble suspensions. The potential flow approximation is applicable in the limits of high bubble Reynolds number and small Weber number $We = \rho V_c^2 \frac{a}{\sigma}$, $\rho$ being the liquid density, $V_c$ the bubble characteristic velocity, $a$ the bubble radius, and $\sigma$ the interfacial tension. It also requires that there be negligible Marangoni effects, so that the tangential stress at the gas-liquid interface is negligible. Comparison between measured rise velocities of gas bubbles in water and theoretical predictions indicate that the potential flow theory for spherical bubbles is reasonably accurate for bubbles of about 1-mm-diameter and the agreement between the theory and experiments can be improved further by including the effects of bubble deformation.\(^3\,^7\)

In addition to the theoretical simplification of the equations of motion at low Reynolds numbers, studies of viscous suspensions have been benefitted from the ability of experimental researchers to isolate and study separately the effects of buoyancy and shearing motion on the suspension structure and rheology. Simple shear flow of a viscous suspension can be obtained by matching the density of the fluid and the particles and/or using a very viscous fluid so that the particles do not settle appreciably during the time of the experiment. While density matching is possible,\(^1\) it cannot be used for bubble suspensions which are most amenable to theoretical treatment.

Equations of motion for bubble suspensions with particular attention to the effects of shearing motion have been developed.\(^4\) For simplicity, bubble-bubble coalescence was neglected. Coalescence can be greatly reduced or avoided without introducing Marangoni effects by using an aqueous electrolyte solution as the continuous phase. Shearing a bubble suspension drives bubble-bubble collisions and the work done to shear the suspension provides a source of kinetic energy for the randomly fluctuating motions of the bubbles. This kinetic energy is dissipated by viscous drag and the balance of shear work and viscous dissipation creates a steady state bubble-phase kinetic energy. The bubbles' random motion and bubble-bubble collisions drive bubbles from regions of high volume fraction to regions of low volume fraction: this phenomenon is described in terms of a bubble-phase pressure. In addition, the bubbles give rise to an effective viscosity that increases the tangential stress required to shear the bubbly liquid. A critical test of these novel, theoretical predictions requires a microgravity experiment in which the effects of shearing can be isolated and measured without the confounding influence of buoyancy driven motion.

Section 2 describes briefly the experiment to be carried out aboard the International Space Station. Section 3 describes the ground-based and low gravity-based research aimed at developing techniques for creating nearly uniform sized, noncoalescing bubble suspensions and measuring bubble volume fraction and velocity distribution. These techniques are being used to study buoyancy driven flow in vertical and inclined channels. Section 4 outlines theoretical framework that will be tested through the experiments.

**THE MICROGRAVITY EXPERIMENT**

Figure 1 shows the schematics of the experiments to be carried out on the International Space Station. A suspension of gas bubbles of approximately 2-mm-diameter in an aqueous solution will be created inside a cylindrical Couette cell. The centrifugal force produced by rotating the outer cylinder will drive the bubbles toward the inner cylinder of the Couette device. However, the shear flow in the gap will create randomly fluctuating bubble velocities and an associated bubble phase pressure. This pressure will resist the accumulation of bubbles and lead to steady state profile of bubble phase volume fraction as a function of
radial position. Hot-film anemometers and dual impedance probes to be described in the next section will be traversed across the gap to determine the bubble volume fraction, velocity, and velocity variance. In addition, a hot film probe mounted flush with the wall will be used to determine the wall shear stress. A comparison of experimentally measured volume fraction, velocity, and velocity variance profiles and wall shear stress with theoretical predictions will provide a critical test of the averaged equations of motion for bubble suspensions and in particular will demonstrate the importance of the bubble phase pressure and viscosity.

Although there is no Taylor-Couette instability when the outer cylinder is rotated, there is still a transition to turbulent flow at significantly high shear rates. This transition will occur in pure water in our Couette design at a shear rate of 27 s⁻¹. We will conduct some of our experiments well below this critical shear rate so that we can be assured of laminar flow. However, an interesting question that the experiment will help us address is whether the effective viscosity produced by the random shear induced bubble velocities will stabilize the flow. Photography of the Couette cell and examination of the spatial and temporal correlations of the probe signals will be used to test for flow instabilities.

GROUND-BASED AND LOW GRAVITY-BASED EXPERIMENTS

Ground-Based Experiments

The ground-based experimental research is focused on (i) producing nearly monodisperse, noncoalescing bubble suspensions; (ii) developing probes for measuring volume fraction and velocity profiles of bubbles; (iii) understanding bubble-wall interactions; and (iv) measurements for flow induced by buoyancy in vertical and inclined channels.

The experimental setup for measurements in vertical channel is shown in Figure 2. The plexiglass cell has a 2-by 20-cm cross-section and the height of 200 cm. Nitrogen gas is introduced at the base of the channel through an array about 100-µm-diameter and 65-mm-long capillaries arranged in a hexagonal array with about 28 capillaries per square cm. The flow rate per capillary used in the experiments is small enough so that the bubble size could be expected to be independent of flow rate. A 0.05

![Diagram of experimental setup](image)
molar solution of MgSO₄ in water was used to inhibit coalescence.

Figure 3 shows photographs of the bubble suspension at volume fractions of 0.05 and 0.10. The bubbles are seen to be nearly monodisperse with an average diameter of about 1.5 mm. Also seen is evidence of clustering of the bubbles in the plane normal to gravity, although the extent of clustering is not as great as seen in the numerical simulations based on potential flow. 11,12,15

Figures 4 and 5 show, respectively, the hot-wire and dual impedance probes used for the measurements. The voltage signals from the dual impedance probes can be autocorrelated to determine the velocity distribution of the bubbles near the probe. The data obtained from the hot-wire can be used to determine the liquid velocity variance and the frequency of collisions between the bubbles and the probe. The latter can be related to the bubble volume fraction and velocity variance.

Representative measurements made with these probes and the setup for vertical channel are shown in Figures 6 to 8. Figure 6 shows the aspect ratio of the bubbles as a function of bubble volume fraction. At larger volume fractions the rise velocity of the bubbles decreases and this causes a decrease in the Weber number, the ratio of inertial to surface tension forces, resulting in bubbles that are less deformed at higher volume fractions. The solid line in the figure corresponds to Moore’s prediction based on an analysis of a single bubble. The bubble phase velocity variance as a function of volume fraction is shown in Figure 7. We see a nearly linear dependence. The results for larger volume fractions extrapolated to zero volume fraction, however, do not give zero variance at zero volume fraction. The wall effects are responsible for this anomalous behavior. 17 Figure 8 shows average rise velocity as a function of bubble volume fraction. Once again the behavior for very dilute bubble suspensions is seen to be different from that obtained by extrapolating higher volume fraction results to small volume fractions.

Low Gravity-Based Experiments

Low gravity-based experiments performed on the KC-135 low gravity facility are focused on the implementation of the diagnostics used for measurement of the local bubble volume fraction and velocity profiles. These experiments are also aimed at the verification of performance of a bread-board Couette system which includes the Couette cell, a two-phase separator, probe diagnostics, and bubble generation subsystem.

The Couette assembly includes a 30-cm-high Couette consisting of a 24-cm-diameter 303 SS stationary inner wall, and 30-cm-diameter acrylic outer wall. The Couette holds approximately 7.6 liters of water between the inner and outer cylinders. Magnesium sulfate (MgSO₄) salt will be added to the water in the Couette to create a 0.05 molar solution to inhibit bubble coalescence. The outer wall is optically clear and is constructed from one piece of acrylic. The outer cylinder rotates from 0 to 100 rpm using a 1/2 HP motor with speed controller. The inner wall consists of 4 segments of the inner cylinder mounted together using 4 splice plates. A (3- by 3-cm) acrylic window is mounted to one of the 4 inner wall segments for imaging the bubbles near the inner wall. The acrylic top and stainless steel bottom of the Couette rotate with the outer cylinder. The bottom includes acrylic windows to provide a light source for illuminating the Couette. The Couette bottom seal material is polymer filled Teflon®. The top uses the same, but a smaller seal design. Two seals provide double containment at all sealing surfaces on the Couette except for the splice plates on the inner cylinder that are equipped with a single gasket seal. The Couette assembly is designed and verified to withstand 7.5 psig. Figure 9 shows a picture of the Couette integrated into the KC-135 rig.
Figure 4.—Schematic of the hot-wire probe and its orientation with respect to the mean bubble motion.

Figure 5.—Schematic of the dual impedance probe. The figure is drawn to scale.
Bubbles are produced in the Couette through 19 capillary tubes (0.02-in.-diameter) spaced equidistant along the height of the Couette and attached to one of the splice plates on the Couette inner cylinder. The 19 bubblers translate in and out of the Couette (0 to 2.5 cm from the inner wall), driven by 5VDC, 4.6W motor. The position of the bubblers relative to the stationary inner wall affects the bubble size produced. An identical motor drives a similar mechanism that supports an impedance and a hot wire probe.

The hot wire probe (used to determine the fluid velocity) and dual impedance probes (used to determine bubble location and velocity) are mounted to the splice plate 90 degrees counter-clockwise from the bubblers and travel from 0 to 2.5 cm across the 3 cm gap between the inner and outer wall of the Couette. This hot wire probe has a conical tip and is expected to be more rugged than previous wire probes. A new signal conditioning unit has been designed and fabricated for the impedance probe circuits.

The separator assembly consists of an acrylic cylindrical chamber that houses 2 concentric acrylic cones with holes machined into the cone sides. The chamber is coupled to a motor that spins the cones at approximately 2500 rpm. When a mixture of gas and liquid enter the separator in the volume between the cones, the liquid is forced to the outside through the holes in the spinning cones into the cylindrical chamber. The gas in the separator forms a distinct core at the center of the inner cone. A gas/liquid detector is mounted in the separator and software control is used for automatic operation. When gas is detected, a solenoid valve opens and the gas is removed by a vacuum pump. When liquid is detected, the solenoid valve closes so that no liquid is removed through the gas vent line. A pump is used to remove the gas/liquid mixture from the Couette test chamber into the separator. The same pump pushes the bubble free water back into the Couette test chamber in preparation for making a new bubble suspension that the researchers are interested in studying. Figure 10 shows a picture of the separator used to separate the gas-liquid mixture in low gravity.

Video cameras (high and normal speeds) are used to study the bubble generation and to assess the bubble size distribution, coalescence, and bubble-probe interactions.

**THEORY**

The equations of motion for large Reynolds number, spherical bubble suspensions have been derived from first principles using theory and numerical simulations. The
Figure 7.—Normalized bubble vertical velocity variance as a function of mean gas volume fraction. The solid squares show the measurements using the dual impedance probe. The dashed and dash-dot lines represent the values of \( A \) used for the bubble velocity predictions for Spelt & Sangani (1998); the dotted line is the fit to the measurements. The circle shows the bubble velocity variance measured for a single bubble in the channel. The diamond represents the bubble velocity variance measured in a very dilute suspension.

Equations consist of two sets: one for the bubble phase and one for the mixture of gas and liquid. The bubble phase equations are

\[
\frac{\partial \phi}{\partial t} + \frac{\partial (\phi V)}{\partial x} = 0
\]  

(1)

\[
\frac{dl}{dt} = -\frac{1}{n} \frac{\partial P}{\partial x} - mg_r - 12\pi \mu_C a V_j
\]

\[+ m \frac{D U_i}{D t} - l_j \frac{\partial U_j}{\partial x_j}
\]  

(2)

where \( \phi \) is the bubble volume fraction, \( t \) the time, \( V \) the average bubble velocity relative to the mixture velocity \( U \), \( l \) the virtual momentum of the bubbles, \( n \) the number density of the bubbles, \( P \) the bubble phase stress tensor, \( m \) the volume of the bubble multiplied by the liquid density, \( g \) the gravitational acceleration, \( a \) the radius of the bubbles, \( C \) the drag coefficient, \( \mu \) the liquid viscosity, \( \frac{d}{dt} \) the time derivative following the motion of bubbles and \( \frac{D}{D t} \) the time derivative following the motion of the gas-liquid mixture. The virtual momentum of the bubbles is given by

\[ l = (m/2) C a V_j, \]

\( C a \) being the virtual mass coefficient. The bubble-phase stress is given by

\[
P_{ij} = \frac{\rho}{2} (1 + 4 \chi \phi) \delta_{ij} - \frac{\rho}{2} G^2 \delta_{ij}
\]

\[+ \left( \kappa - \frac{2\mu_s}{\chi} \right) e_{ik} \delta_{ij} - 2\mu_s e_{ij}
\]

(3)

where \( T \) is the bubble phase temperature, defined as one-third of the bubble velocity variance, \( \chi \) the radial distribution function value at \( 2a \), \( \kappa \) and \( \mu_s \) the bubble-phase viscosities, \( G = (\text{Ca}/2 + 1) V_j \), and \( e_{ij} \) the rate of strain tensor for the bubble phase. The isotropic part of the bubble phase stress is the bubble phase pressure. The bubble-phase stress and the other properties such as the bubble-phase viscosities depend on the bubble phase temperature and therefore an additional equation is required for the balance of bubble fluctuation energy. This equation, the continuity and momentum equations for the mixture, and the closure relations for the bubble phase viscosity.
Figure 8.—Bubble velocity as a function of mean gas volume fraction. The solid squares show the measurements using the dual impedance probe. The lines show the predictions from Spelt & Sangani (1998) using $u_\infty = 0.320$ m s$^{-1}$ and different values of the parameter $A$: $A = 10$; $A = 20$; $A = A(\alpha)$ from experiments. The filled circle shows the terminal velocity of a bubble moving in a large channel ($u_\infty = 0.320$ m s$^{-1}$). The empty circle shows the velocity measured for a single bubble in the experimental channel. The diamond shows the measurement for a very dilute suspension in the channel.

Figure 9.—Picture of the Couette system hardware.
Figure 10.—Photos showing the performance of the separator, (a) in the beginning (b) the middle, and (c) toward the end of the separation process. Note the gas core forming in the middle.


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