CHAPTER 1

AN ELLIPSOIDAL PARTICLE-FINITE ELEMENT METHOD FOR HYPERVELOCITY IMPACT SIMULATION

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A number of coupled particle-element and hybrid particle-element methods have been developed for the simulation of hypervelocity impact problems, to avoid certain disadvantages associated with the use of pure continuum based or pure particle based methods. To date these methods have employed spherical particles. In recent work a hybrid formulation has been extended to the ellipsoidal particle case. A model formulation approach based on Lagrange’s equations, with particles entropies serving as generalized coordinates, avoids the angular momentum conservation problems which have been reported with ellipsoidal smooth particle hydrodynamics models.

KEYWORDS: particle methods, finite element methods, impact simulation

INTRODUCTION

A review of the literature on hypervelocity impact simulation shows that most research in this field has focused on the development and application of continuum hydrocodes, using either Eulerian, Lagrangian, or Arbitrary Lagrangian-Eulerian (ALE) formulations [1,2,3]. In the last decade attention has shifted towards particle based methods [4], in particular smooth particle hydrodynamics (SPH). Although both continuum and particle based methods have demonstrated excellent results in a range of practical applications, these methods are not without problems. As a result some recent research has formulated coupled particle-element

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[5] or hybrid particle-element [6] methods, aimed at avoiding certain disadvantages of the use of pure continuum based methods or pure particle based methods in hypervelocity impact applications [7]. The present paper develops a generalized hybrid method, extending the work of Fahrenthold and Horban [6], presenting for the first time a combined particle-element formulation based on a nonspherical particle geometry. Nonspherical particle formulations are also of interest in certain molecular dynamics [8] and astrophysics [9,10] applications, where material or flowfield geometry has suggested the use of repulsion potentials or interpolation kernels with ellipsoidal forms.

Most previous hypervelocity impact simulation work has employed Eulerian finite volume and Lagrangian finite element methods [1,11] and more recent extensions of these methods to Arbitrary Lagrangian-Eulerian frames [12]. Although continuum hydrocodes are accurate and efficient simulation tools for many important problems, they are not well suited for use in all hypervelocity impact applications. Eulerian codes can accommodate arbitrarily general contact-impact, but their approximate models of material strength effects are best suited to a very high velocity impact regime. Lagrangian codes incorporate very accurate models of material strength effects, but their slideline based contact-impact algorithms are best suited to a relatively low velocity impact regime. ALE methods allow for intelligent tailoring of the mesh, ideally avoiding (but not eliminating) the preceding limitations.

None of the aforementioned continuum formulations is ideally suited to applications where the generation, transport, and contact-impact dynamics of material fragments is of central concern. When material fragments become much smaller than the finite volume cell size, Eulerian flow field interpolations may not accurately represent the physical state of the highly comminuted medium. Hence the adaptive introduction of a high resolution Eulerian mesh is required. Similarly the use of Lagrangian finite elements to represent a highly fragmented medium calls for the introduction of a very extensive and adaptive slideline mesh. It is not surprising that these continuum formulations can be difficult to apply in cases where fragmentation dynamics are of central interest, given the highly discontinuous nature of the
field variables in such systems.

Recognizing the disadvantages of continuum formulations in certain hypervelocity impact applications, a number of different particle methods [13,14,15] have been introduced. In general these methods replace or augment the conventional continuum kinematics with particle based interpolations, to more efficiently represent fragment generation, transport, and contact-impact effects in applications where modeling of the latter physics is of central concern. The overwhelming majority of particle based hypervelocity impact models have employed SPH techniques and spherical kernels. A notable exception is the work of Shapiro and co-workers [9,10], who extended the basic SPH method to ellipsoidal kernels. However those authors indicate that their method fails to conserve angular momentum, and they offer no solutions to the tensile instability, numerical fracture, and other problems which have hindered the effective use of SPH methods in some applications. The formulation presented here does not involve element-to-particle conversion, as discussed for example by Johnson et al. [14], who converted distorted finite elements into SPH particles. The latter conversion, if it occurs before failure of the element, may introduce the aforementioned SPH tensile instability and and numerical fracture problems.

The present paper formulates a hybrid ellipsoidal particle-finite element model for hypervelocity impact simulation. Unlike the coupled particle-element formulations of Hayhurst et al. [16] and other authors, which use particles to model some structures and finite elements to model other structures, the present paper employs particles and elements everywhere. The simultaneous use of particles and elements is not redundant, since they are used to represent distinct physics. The particles model all inertia and contact-impact effects, and are associated with interpolation functions which represent compressed states. The elements model all strength effects, namely tension and elastic-plastic shear. This approach has several advantages. The tensile instability and numerical fracture problems common to SPH formulations are avoided, since interpolation kernels are not used to represent interparticle tension forces, and since material strength effects give rise to interparticle forces only among reference co-
configuration neighbors. Although true Lagrangian kinematics are used to calculate tensile and shear forces, the use of particles in all inertia and contact-impact calculations means that material failure is simply accommodated (without element-to-particle remapping or mass and energy discard) via the loss of element cohesion. Particles not associated with any intact elements translate and rotate as individual fragments, in response to contact-impact loads. No slidelines are used, so that contact-impact of all intact and fragmented material is modeled everywhere. Although the authors do not contend that this numerical approach is best for all impact simulation problems, it offers important advantages in hypervelocity impact applications. In the latter field an interest in multi-structure perforation, erosion, fragmentation, and very general contact-impact modeling is not unusual.

The model formulation work described in the sections which follow is different from that normally employed to develop hydrocode models of either the continuum or particle type. Two particular features should be mentioned. First an energy method (Lagrange’s equations) is used to develop the semi-discrete model, so that no reference is made to any partial differential equations. The introduction of entropy variables as generalized coordinates makes it possible to account for very general thermal dynamics while using a discrete Lagrangian approach. Secondly the formulation introduces Euler parameter states [17] to account for the rotational motion of the nonspherical particles. An Euler parameter description of the particle kinematics is desired in order to avoid the singularities of Euler angle based models. Although the use of Euler parameters leads nominally to a differential-algebraic formulation, the development which follows introduces momentum state variables, to reduce the order of the system and thereby eliminate the Lagrange multipliers associated with the Euler parameter constraint. The model formulation work described here therefore generalizes in a geometric sense the particle-element work of Fahrenthold and Horban [6] and generalizes in a thermomechanical sense the Euler parameter based mechanical models of Chang and Chou [18] and others [19,20].

The organization of this paper may be summarized as follows. The second and third
sections define the particle and element kinematics and the interpolations used for all field variables. The fourth and fifth sections develop kinetic energy and thermomechanical potential energy functions for the system. The sixth section describes the dissipative process models, for plastic formation and damage evolution. The seventh and eighth sections present artificial viscosity and artificial heat diffusion models, similar to those used in other shock physics codes, and develop entropy evolution equations which take the place of the energy conservation relations normally employed. The ninth section introduces a virtual work expression, to account for externally applied loads. The tenth section combines the canonical thermomechanical Lagrange equations with the multiple nonholonomic constraints developed in the preceding sections, and derives an unconstrained explicit state space model for the particle-element system. The final section presents two example problems, showing good agreement of simulations performed using the model developed here to published data for three dimensional impact experiments. Additional validation work, for simulations performed using a spherical kernel, are described by Fahrenthold and Shivarama [21], who also provide details on the numerical implementation and test results showing numerical convergence and good parallel speedup.

PARTICLE KINEMATICS

The inertia distribution in the modeled system is represented by a collection of \( n \) ellipsoidal particles, with \( m^{(i)} \) the mass of the \( i \)th particle, whose position and orientation are described by a center of mass position vector \( (c^{(i)}) \) and an Euler parameter vector \( (e^{(i)}) \) with component forms

\[
\begin{align*}
(c^{(i)}) &= [c_1^{(i)} c_2^{(i)} c_3^{(i)}]^T, \\
(e^{(i)}) &= [e_0^{(i)} e_1^{(i)} e_2^{(i)} e_3^{(i)}]^T
\end{align*}
\]

(1)

where \( T \) denotes the transpose. The components of the center of mass position vector are described in a fixed Cartesian coordinate system. The Euler parameters [17] define an orthogonal rotation matrix \( (R^{(i)}) \) for each particle

\[
R^{(i)} = H^{(i)} G^{(i)T}
\]

(2)
\[ H^{(i)} = \begin{bmatrix} -e_1^{(i)} & e_0^{(i)} & -e_3^{(i)} & e_2^{(i)} \\ -e_2^{(i)} & e_3^{(i)} & e_0^{(i)} & -e_1^{(i)} \\ -e_3^{(i)} & -e_2^{(i)} & e_1^{(i)} & e_0^{(i)} \end{bmatrix} \]

(3)

\[ G^{(i)} = \begin{bmatrix} -e_1^{(i)} & e_0^{(i)} & e_3^{(i)} & -e_2^{(i)} \\ -e_2^{(i)} & -e_3^{(i)} & e_0^{(i)} & e_1^{(i)} \\ -e_3^{(i)} & e_2^{(i)} & -e_1^{(i)} & e_0^{(i)} \end{bmatrix} \]

(4)

which relates the components of any vector \((\hat{a})\), described in a body fixed co-rotating coordinate system, to its corresponding components in the fixed Cartesian system, using

\[ a = R^{(i)} \hat{a} \]

(5)

The four Euler parameters may be used to compute three Euler angles for each particle, and are preferred for use in large rotation dynamics problems, since the Euler angles are singular functions. Although the Euler parameters are nonsingular, they must satisfy the constraint

\[ e^{(i)T} e^{(i)} = 1 \]

(6)

which requires that

\[ G^{(i)} e^{(i)} = 0, \quad G^{(i)} \dot{e}^{(i)} = -\dot{G}^{(i)} e^{(i)}, \quad G^{(i)} G^{(i)T} = I \]

(7)

where \( I \) denotes an identity matrix.

The Euler parameters and their time derivatives are related to the particle angular velocity vectors \((\omega^{(i)})\), with components expressed in the particle’s co-rotating frame, by the well known [17] relations

\[ \omega^{(i)} = 2 G^{(i)} \dot{e}^{(i)}, \quad \dot{e}^{(i)} = \frac{1}{2} G^{(i)T} \omega^{(i)} \]

(8)

Similarly the anti-symmetric matrix \(\Omega\), with axial vector \(\omega\), which satisfies

\[ \Omega^{(i)} v = \omega^{(i)} \times v \]

(9)

for any vector \(v\), is related to the Euler parameters and their time derivatives by

\[ \Omega^{(i)} = 2 G^{(i)} \dot{G}^{(i)T} = -2 \dot{G}^{(i)} G^{(i)T} \]

(10)
The next section presents a density interpolation associated with the particles just described, and defines Lagrangian finite elements whose nodal coordinates are components of the particle center of mass position vectors.

### DENSITY INTERPOLATION AND FINITE ELEMENTS

In the material reference configuration, the ellipsoidal particles are arranged in a packing scheme defined by a mapping from a body centered cubic unit cell, the mapping composed of stretchings in three orthogonal directions, those directions aligned with the principal axes of the ellipsoids. The density interpolation for compressed states is

\[
\rho^{(i)} = \rho_o^{(i)} + \sum_{j=1}^{n} \rho_o^{(j)} W^{(i,j)}
\]  

(11)

where \(\rho^{(i)}\) is the continuum density at the centroid of the \(i\)th particle, \(\rho_o^{(i)}\) is the reference density for the \(i\)th particle, and \(W^{(i,j)}\) is the interpolation kernel

\[
W^{(i,j)} = \frac{1}{8} (1 - \delta_{ij}) \left[ \left( \frac{1}{\xi^{(i,j)}} \right)^3 - 1 \right] \hat{u} \left( \xi^{(i,j)} \right)
\]

(12)

\[
\xi^{(i,j)} = \left( \frac{\alpha + \beta}{2} \right) \left( \frac{\rho_o^{(i)}}{\rho^{(i)}} \right)^{\frac{1}{3}} - \zeta^{(i,j)}
\]

(13)

The symbols \(\delta_{ij}\) and \(\hat{u}\) denote respectively the Dirac delta function and the unit step function

\[
\hat{u}(x) = \begin{cases} 
0, & x \leq 0 \\
1, & x > 0 
\end{cases}
\]

(14)

The constants \(\alpha\) and \(\beta\) are effective separation distances, in a unit cell, measured respectively between body centered particles and between a body centered particle and a particle located at a cell vertex

\[
\alpha = \left( \frac{\pi}{3} \right)^{\frac{1}{3}}, \quad \beta = \frac{\sqrt{3}}{2} \left( \frac{\pi}{3} \right)^{\frac{1}{3}}
\]

(15)

Note that the lead coefficient in the expression for \(W^{(i,j)}\) is due to the presence of eight nearest neighbors in the reference configuration.
The function $\zeta^{(i,j)}$ is an ellipsoidal coordinate, defined in a frame which co-rotates with the $j$th particle

$$\zeta^{(i,j)} = \left[ (\mathbf{c}^{(i)} - \mathbf{c}^{(j)})^T \mathbf{R}^{(j)} \mathbf{A}^{(j)} \mathbf{R}^{(j)^T} (\mathbf{c}^{(i)} - \mathbf{c}^{(j)}) \right]^{\frac{1}{2}}$$

(16)

$$\mathbf{A}^{(j)} = \begin{bmatrix} 2\beta h_1^{(j)} & 0 & 0 \\ 0 & 2\beta h_2^{(j)} & 0 \\ 0 & 0 & 2\beta h_3^{(j)} \end{bmatrix}^{-2}$$

(17)

where $h_1^{(j)}, h_2^{(j)}, h_3^{(j)}$ are the half-lengths of the principal axes of the $j$th particle.

Several properties of the density interpolation and kernel functions just described should be noted. The interpolation provides an exact Lagrangian description of the variation of density under uniform compression, and incorporates a lower bound equal to the particle reference density, so that tensile instabilities are avoided. The kernel is singular, so that no two particle centers overlap, avoiding particle streaming effects. In the reference configuration neighbor particles do not contribute to the density calculation, so that no special treatment of surface particles with incomplete neighbor sets is required. The density dependence of the compact kernel support means that particles interact only with near neighbors, and that mechanical interaction with remote neighbors through intervening matter is avoided. Finally note that the dimensionless kernel used here is closer in functional form and physical interpretation to the repulsion potentials used in molecular dynamics, than to the dimensional kernels with cubic spline form used in most SPH formulations.

The center of mass coordinates of vertex centered particles are nodal coordinates for hexahedral finite elements, which are used to account for tension and elastic-plastic shear in cohesive solid materials. These elements are used with one point integration, as described by Hallquist [22], although in the present case the element packing scheme and density interpolation preclude the development of hourglass modes. Since large deformation problems are of interest in hypervelocity impact applications, the element nodal coordinates are used here to compute a Jacobian ($J^{(j)}$) and a Lagrangian deviatoric strain tensor ($\mathbf{E}^{(j)}$) for each
element, defined by [23]

\[ J^{(j)} = \det \mathbf{F}^{(j)}, \quad \mathbf{E}^{(j)} = \frac{1}{2} \left( \mathbf{C}^{(j)} - \mathbf{I} \right) \]  

(18)

where \( \mathbf{F}^{(j)} \) is the deformation gradient for the \( j \)th element and

\[ \mathbf{C}^{(j)} = \mathbf{F}^{(j)T} \mathbf{F}^{(j)}, \quad \mathbf{F}^{(j)} = \left( \det \mathbf{F}^{(j)} \right)^{-\frac{1}{3}} \mathbf{F}^{(j)} \]  

(19)

The plasticity model discussed in a later section assumes an additive decomposition of the deviatoric strain, with the elastic deviatoric strain tensor (\( \mathbf{E}^{e(j)} \)) for the \( j \)th element defined by

\[ \mathbf{E}^{e(j)} = \mathbf{E}^{(j)} - \mathbf{E}^{p(j)} \]  

(20)

so that the evolution equations for the plastic strain tensor (\( \mathbf{E}^{p(j)} \)) must satisfy the isochoric plastic deformation constraint

\[ \text{tr} \left( \mathbf{C}^{p(j)-T} \dot{\mathbf{C}}^{p(j)} \right) = 0, \quad \mathbf{E}^{p(j)} = \frac{1}{2} \left( \mathbf{C}^{p(j)} - \mathbf{I} \right) \]  

(21)

It should be emphasized that the numerical modeling methodology developed in this paper applies for a wide range of element types and plasticity models. The particular element kinematics and constitutive equations used in this paper are representative formulations which account for large deformation kinematics.

**KINETIC ENERGY**

The kinetic co-energy for the system is

\[ T^* = \sum_{i=1}^{n} T^{*(i)} \]  

(22)

where \( T^{*(i)} \) is the kinetic co-energy for the \( i \)th particle

\[ T^{*(i)} = \frac{1}{2} m^{(i)} \dot{\mathbf{c}}^{(i)} T \dot{\mathbf{c}}^{(i)} + \frac{1}{2} \omega^{(i)} T \mathbf{J}^{(i)} \omega^{(i)} \]  

(23)

and the components of the inertia matrix \( \mathbf{J}^{(j)} \) are calculated in a body fixed frame. Since the components of the angular velocity vector are quasi-velocities, equations (8) are used to
rewrite the second term

\[ T^{*\text{(i)}} = \frac{1}{2} m^{(i)} \dot{c}^{(i)} T \dot{c}^{(i)} + 2 \dot{\epsilon}^{(i)} T G^{(i)} T J^{(i)} G^{(i)} \dot{\epsilon}^{(i)} \]  

(24)

which identifies the center of mass coordinates and the Euler parameters as generalized coordinates in Lagrange’s equations. Note that the relations (7) allow the rotational kinetic co-energy to be expressed as

\[ T_{\text{rot}}^{\text{*\text{(i)}}} = 2 \epsilon^{(i)} T \ddot{G}^{(i)} T J^{(i)} \dot{G}^{(i)} e^{(i)} \]  

(25)

so that the generalized force associated with the Euler parameter dependence of the kinetic co-energy is

\[ k^{(i)} = \frac{\partial T_{\text{rot}}^{\text{*\text{(i)}}}}{\partial e^{(i)}} = 4 \ddot{G}^{(i)} T J^{(i)} \dot{G}^{(i)} e^{(i)} \]  

(26)

The next section develops a potential energy expression for the particle-element system.

**POTENTIAL ENERGY**

The potential energy of the particle-element system is the sum of the particle internal energies and the element stored energies due to tension and elastic shear, and has the general form

\[ V = \sum_{i=1}^{n} m^{(i)} u^{(i)} + \sum_{j=1}^{n_e} \left( U^{\text{ten}(j)} + U^{\text{dev}(j)} + U^{\text{pen}(j)} \right) \]  

(27)

where \( u^{(i)} \) is an internal energy per unit mass, \( n_e \) is the number of elements, \( U^{\text{ten}(j)} \) is a stored energy in tension, \( U^{\text{dev}(j)} \) is a stored energy in shear, and \( U^{\text{pen}(j)} \) is a penalty energy used to position the body centered particle in each element. The particle internal energy density is determined by the mass density and entropy density \( (s^{(i)}) \)

\[ u^{(i)} = u^{(i)} \left( \rho^{(i)}, s^{(i)} \right), \quad s^{(i)} = \frac{S^{(i)}}{m^{(i)}} \]  

(28)

where \( S^{(i)} \) is the total entropy of the \( i \)th particle. The functional form of the internal energy per unit mass depends on the equation of state, and it is not unusual in such calculations to
rely upon tabular data [24]. In any case, the extensive state variables which determine the particle internal energy are

\[
    u^{(i)} = u^{(i)}(c^{(j)}, e^{(j)}, S^{(j)})
\]

(29)

The form of the element stored energy functions depends upon the constitutive assumptions. Although the general method developed here admits a wide range of constitutive models, simple forms are assumed here, and are described as follows. The stored energy in tension is

\[
    U^{ten(j)} = \frac{1}{2} (1 - D^{(j)}) \kappa^{(j)} V_o^{(j)} (J^{(j)} - 1)^2 \dot{u}(J^{(j)} - 1)
\]

(30)

where \(\kappa^{(j)}\) is a bulk modulus, \(D^{(j)}\) is the normal damage, and \(V_o^{(j)}\) is the element reference volume. The stored energy in shear is

\[
    U^{dev(j)} = (1 - d^{(j)}) \mu^{(j)} V_o^{(j)} tr \left( E_e^{(j)} T E_e^{(j)} \right)
\]

(31)

where \(\mu^{(j)}\) is a shear modulus and \(d^{(j)}\) is the deviatoric damage. The penalty function is

\[
    U^{pen(j)} = \frac{1}{2} (1 - D^{(j)}) K^{(j)} (\bar{c}^{(j)} - \bar{c}^{(j)})^2
\]

(32)

where \(\bar{c}^{(j)}\) is the center of mass position vector for the body centered particle of the \(j\)th element, \(\bar{c}^{(j)}\) is the average of the center of mass position vectors for the vertex centered particles (the element nodes), and the penalty stiffness is

\[
    K^{(j)} = E^{(j)} V_o^{(j)} \frac{1}{3}
\]

(33)

where \(E^{(j)}\) is Young’s modulus for the element. The damage variables are used to model the loss of cohesion due to element failure, and are discussed in the next section.

The preceding constitutive assumptions combine with the element kinematic relations

\[
    J^{(j)} = J^{(j)}(c^{(i)}), \quad E_e^{(j)} = E_e^{(j)}(c^{(i)}, E_p^{(j)})
\]

(34)

to yield the following state variable dependence for the system potential energy

\[
    V = V(c^{(i)}, e^{(i)}, S^{(i)}, E_p^{(j)}, D^{(j)}, d^{(j)})
\]

(35)
The listed arguments, which include the particle entropies, are by definition Lagrangian generalized coordinates.

The system potential energy function defines the conservative generalized forces

$$\frac{\partial V}{\partial c(i)} = g(i), \quad \frac{\partial V}{\partial e(i)} = M(i), \quad \frac{\partial V}{\partial S(i)} = \theta(i)$$ (36)

where $\theta(i)$ is a particle temperature, and $g(i)$ and $M(i)$ are forces and torques which act on the ellipsoidal particles. The energy conjugates for the dissipative state variables define a deviatoric stress

$$S(j) = -\frac{1}{V_0} \frac{\partial V}{\partial E_p(j)}$$ (37)

as well as the energy release rates

$$\Gamma^D(j) = -\frac{\partial V}{\partial D(j)}, \quad \Gamma^d(j) = -\frac{\partial V}{\partial d(j)}$$ (38)

The next section discusses evolution equations for the dissipative state variables.

**PLASTICITY AND DAMAGE MODELS**

The evolution equations for the plastic strain tensor and continuum damage variables will serve as nonholonomic constraints on the system level Lagrange equations. As discussed in the last section, the general formulation developed here admits a wide range of constitutive models. The relatively simple dissipative constitutive models assumed here are described in this section. The plasticity model is adapted from Fahrenthold and Horban [25], and represents the simplest possible accommodation of the isochoric deformation constraint (21). The damage evolution relations are adapted from the Eulerian hydrocode work of Silling [26].

The flow rule for the plastic strain is

$$\dot{E}^{p(j)} = \dot{\lambda}^{(j)} \frac{\Pi^{(j)}}{\Pi^{(j)}} (C^{p(j)} W^{(j)} + W^{(j)} C^{p(j)})$$ (39)

$$W^{(j)} = C^{p(j)} S^{p(j)} + S^{p(j)} C^{p(j)} - \frac{1}{3} tr (C^{p(j)} S^{p(j)} + S^{p(j)} C^{p(j)}) I$$ (40)

where $\dot{\lambda}^{(j)}$ is a scalar multiplier and

$$\Pi^{(j)} = (1 + \eta^{(j)} \epsilon^{p(j)}) N^{(j)} \left[ \frac{1}{2} tr (W^{(j)T} W^{(j)}) \right]^{1/2}$$ (41)
\[ \varepsilon^p(j) = \left[ \frac{1}{2} \text{tr} \left( \dot{E}^p(j)^T \dot{E}^p(j) \right) \right]^{1/2} \]

with \( \eta^{(j)} \) a strain hardening modulus, \( N^{(j)} \) a strain hardening exponent, and \( \varepsilon^p(j) \) the accumulated plastic strain. The effective stress tensor which appears in the flow rule is

\[ S^p(j) = (1 + \eta^{(j)} \varepsilon^p(j))^{-N^{(j)}} S^{(j)}, \quad S^{(j)} = -\frac{1}{V^{(j)}} \frac{\partial V}{\partial \dot{E}^p(j)} = (1 - d^{(j)}) 2 \mu^{(j)} \dot{E}^e(j) \]

where the indicated deviatoric stress is power density conjugate to the plastic strain rate, in the entropy equality for the solid medium.

The yield condition is

\[ f^{(j)} = \tau^{(j)} - Y^{(j)}, \quad \tau^{(j)} = \left[ \frac{1}{2} \text{tr} \left( S^p(j)^T S^p(j) \right) \right]^{1/2} \]

where \( \tau^{(j)} \) is the second invariant of the effective stress. The yield stress \( Y \) is

\[ Y^{(j)} = Y_0^{(j)} (1 - d^{(j)}) \left( 1 - \gamma^{(j)} \theta^{H(j)} \right) \]

where \( Y_0^{(j)} \) is the reference yield stress, \( \gamma^{(j)} \) is a thermal softening modulus, and \( \theta^{H(j)} \) is the maximum historical homologous temperature. The plastic strain increment at each time step is determined using a one step iteration procedure [22] with

\[ \Delta \lambda^{(j)} = \frac{(\tau^{(j)} - Y^{(j)}) \dot{u}(\tau^{(j)} - Y^{(j)})}{(1 - d^{(j)}) 2 \mu^{(j)}} \]

The final evolution equations for the plastic strain have the functional form

\[ \dot{E}^p(j) = \dot{E}^p(j) \left( \epsilon^{(i)}, S^{(i)}, E^p(j), d^{(i)} \right) \]

and are nonholonomic constraints on the particle-element model.

Hypervelocity impact problems normally involve perforation and fragmentation effects, so that the ability to model such physics is essential. In the present case normal and deviatoric damage variables are introduced, to model the transition from a cohesive solid, characterized by intact finite elements, to a comminuted medium, described by the free flow of particles.
Free particles interact with cohesive material and with other fragments under general contact-impact loads.

The damage evolutions applied here are

$$\dot{D}^{(j)} = \frac{\Lambda^{(j)}}{\tilde{n}} \Delta t \hat{u}(1 - D^{(j)}), \quad \dot{d}^{(j)} = \frac{\Lambda^{(j)}}{\tilde{n}} \Delta t \hat{u}(1 - d^{(j)})$$

(48)

where $\Delta t$ is the time step, $\tilde{n}$ is the number of time steps used to model the transition from an intact to a failed element, and

$$\Lambda^{(j)} = \max\{ \hat{u}(\epsilon_p^{(j)} - \epsilon_f^{(j)}), \hat{u}(\theta_{\max}^{(j)} - \theta_m^{(j)}), \hat{u}(J_c^{(j)} - J_{\min}^{(j)}), \hat{u}(\sigma_{\max}^{(j)} - \sigma_s^{(j)}) \}$$

(49)

The function just defined initiates damage evolution in an element when when the accumulated plastic strain $\epsilon_p^{(j)}$, maximum temperature $\theta_{\max}^{(j)}$, minimum element Jacobian $J_{\min}^{(j)}$, or maximum eigenvalue of the deviatoric stress $\sigma_{\max}^{(j)}$ reach corresponding critical values for the failure strain ($\epsilon_f^{(j)}$), melt temperature ($\theta_m^{(j)}$), maximum compression ($J_c^{(j)}$), or spall stress ($\sigma_s^{(j)}$). More sophisticated damage evolution relations may be introduced without change to the general modeling framework. The damage evolution equations are nonholonomic constraints which apply to the system level Lagrange equations.

**ARTIFICIAL VISCOSITY AND HEAT DIFFUSION**

The shock physics problems of interest here call for the introduction of artificial viscosity and artificial heat diffusion effects. The standard formulation used here takes the viscosity force $f^{v(i)}$ on the $i$th particle to be

$$f^{v(i)} = \sum_{j=1}^{n} \nu^{(i,j)} \max(0, v^{(i,j)}) \ r^{(i,j)}, \quad r^{(i,j)} = \frac{c^{(i)} - c^{(j)}}{|c^{(i)} - c^{(j)}|}$$

(50)

where the velocity difference $v^{(i,j)}$ is positive for converging particles

$$v^{(i,j)} = - (\dot{c}^{(i)} - \dot{c}^{(j)}) \cdot r^{(i,j)}$$

(51)

The damping coefficient $\nu^{(i,j)}$ is

$$\nu^{(i,j)} = \frac{1}{2} \left( \rho^{(i)} c_s^{(i)} V_o^{(i)} \frac{\hat{v}^{(i)}}{\hat{v}} + \rho^{(j)} c_s^{(j)} V_o^{(j)} \frac{\hat{v}^{(j)}}{\hat{v}} \right) \left[ c_0 + \frac{2 c_1 |v^{(i,j)}|}{(c_s^{(i)} + c_s^{(j)})} \right] \hat{u}(1 - \zeta^{(i,j)})$$

(52)
where $c_s^{(i)}$ is the sound speed for the $i$th particle and the parameters $c_o$ and $c_1$ are dimensionless linear and quadratic numerical viscosity coefficients.

The numerical heat diffusion model assumed here takes the thermal power flow for the $i$th particle to be

$$
\dot{Q}_{\text{con}}^{(i)} = \sum_{j=1}^{n} R^{(i,j)} (\theta^{(i)} - \theta^{(j)})
$$

(53)

$$
R^{(i,j)} = \frac{1}{2} \left( \rho^{(i)} c_s^{(i)} c_v^{(i)} V_o^{(i)^{\frac{3}{2}}} + \rho^{(j)} c_s^{(j)} c_v^{(j)} V_o^{(j)^{\frac{3}{2}}} \right) k_o \hat{u} (1 - \zeta^{(i,j)})
$$

(54)

where $c_v^{(i)}$ is the specific heat for the $i$th particle and the parameter $k_o$ is a dimensionless numerical heat diffusion coefficient.

**ENTROPY EVOLUTION EQUATIONS**

The introduction of entropy state variables, which facilitates the use of an energy based modeling approach, means that the conservation of energy relations normally incorporated in shock physics models are replaced here by evolution equations for the particle entropies. These evolution equations are

$$
\dot{S}^{(i)} = \dot{S}^{\text{irr}(i)} - \dot{S}^{\text{con}(i)}
$$

(55)

where the irreversible entropy production ($\dot{S}^{\text{irr}(i)}$) is due to plastic deformation, damage evolution, and viscous dissipation

$$
\dot{S}^{\text{irr}(i)} = \theta^{(i)-1} \left[ \Gamma^{v^{(i)}} T^{(i)} e^{(i)} + \sum_{j=1}^{n_e} \phi^{(i,j)} \left\{ \Gamma^{D^{(j)}} \dot{D}^{(j)} + \Gamma^{d^{(j)}} \dot{d}^{(j)} + V_o^{(j)} tr \left( S^{(j)} T^{\text{E}^{p(j)}} \right) \right\} \right]
$$

(56)

and $\phi^{(i,j)}$ is the fraction of the dissipation in element $j$ associated with the $i$th particle. The conduction entropy flow ($\dot{S}^{\text{con}(i)}$) is due to artificial heat diffusion

$$
\dot{S}^{\text{con}(i)} = \theta^{(i)-1} \dot{Q}^{\text{con}(i)} = \sum_{j=1}^{n} R^{(i,j)} (1 - \frac{\theta^{(j)}}{\theta^{(i)}})
$$

(57)

Since the dissipated energy is not lost but rather transduced to the thermal domain, the entropy evolution equations are nonholonomic constraints in the thermomechanical Lagrangian formulation.
VIRTUAL WORK

Although external loads are normally not considered in hypervelocity impact applications, for completeness this section develops a virtual work expression. The virtual work for the system is a summation over the particles

$$\delta W = \sum_{i=1}^{n} \delta W^{(i)}$$

(58)

In this case the particle angular velocities are the time derivatives of rotational quasi-coordinates ($q^{(i)}$)

$$\dot{q}^{(i)} = \omega^{(i)}$$

(59)

so that the virtual work for the $i$th particle is

$$\delta W^{(i)} = f^{(i)T} \delta c^{(i)} + T^{(i)T} \delta q^{(i)}$$

(60)

where $f^{(i)}$ and $T^{(i)}$ are externally applied forces and torques. In view of equation (8), virtual changes in the quasi-coordinates and the Euler parameters are related by

$$\delta q^{(i)} = 2 G^{(i)} \delta e^{(i)}$$

(61)

so that the particle virtual work expression is

$$\delta W^{(i)} = f^{(i)T} \delta c^{(i)} + 2 \left[ G^{(i)T} T^{(i)} \right]^T \delta e^{(i)}$$

(62)

The coefficients of the virtual changes in the generalized coordinates are by definition generalized nonconservative forces in the system level Lagrange equations.

LAGRANGE’S EQUATIONS

The stored energy functions, constraint equations, and virtual work expression developed in the preceding sections may be combined with the canonical Lagrange equations, to obtain an ODE model for the particle-element system.
The canonical Lagrange equations are

\[
\frac{d}{dt} \left( \frac{\partial T^*}{\partial \dot{c}^{(i)}} \right) - \frac{\partial T^*}{\partial c^{(i)}} + \frac{\partial V}{\partial c^{(i)}} = Q^{c(i)} \tag{63}
\]

\[
\frac{d}{dt} \left( \frac{\partial T^*}{\partial \dot{e}^{(i)}} \right) - \frac{\partial T^*}{\partial e^{(i)}} + \frac{\partial V}{\partial e^{(i)}} = Q^{e(i)} \tag{64}
\]

\[
\frac{\partial V}{\partial S^{(i)}} = Q^{s(i)}, \quad \frac{\partial V}{\partial \Gamma^{p(j)}} = Q^{p(j)} \tag{65}
\]

\[
\frac{\partial V}{\partial d^{(j)}} = Q^{d(j)}, \quad \frac{\partial V}{\partial D^{(j)}} = Q^{D(j)} \tag{66}
\]

where \(Q^{c(i)}, Q^{e(i)}, Q^{s(i)}, Q^{d(i)}, Q^{D(i)},\) and \(Q^{p(j)}\) are generalized forces determined by the constraints and the virtual work. Note that the rate form of the Euler parameter constraint for the \(i\)th particle is

\[
\dot{e}^{(i)T} e^{(i)} = 0 \tag{67}
\]

Introducing a Lagrange multiplier \(\gamma^{e(i)}\) for each Euler parameter constraint as well as Lagrange multipliers \(\gamma^{s(i)}, \gamma^{d(i)}, \gamma^{D(i)},\) and \(X^{p(i)}\) for the entropy, normal damage, deviatoric damage, and plastic strain evolutions equations, the generalized nonconservative forces are found to be

\[
Q^{c(i)} = f^{(i)} - \frac{\gamma^{s(i)}}{\theta^{(i)}} f^{v(i)} \tag{68}
\]

\[
Q^{e(i)} = 2 G^{(i)T} T^{(i)} + \gamma^{e(i)} e^{(i)} \tag{69}
\]

\[
Q^{s(i)} = \gamma^{s(i)} \tag{70}
\]

\[
Q^{d(j)} = \gamma^{d(j)} - \sum_{i=1}^{n} \frac{\gamma^{s(i)}}{\theta^{(i)}} \phi^{(i,j)} \Gamma^{d(j)} \tag{71}
\]

\[
Q^{D(j)} = \gamma^{D(j)} - \sum_{i=1}^{n} \frac{\gamma^{s(i)}}{\theta^{(i)}} \phi^{(i,j)} \Gamma^{D(j)} \tag{72}
\]

\[
Q^{p(j)} = X^{p(j)} - \sum_{i=1}^{n} \frac{\gamma^{s(i)}}{\theta^{(i)}} \phi^{(i,j)} V^{(j)} S^{(j)} \tag{73}
\]
If these results and the degenerate Lagrange equations (65) and (66) are combined with the previously derived expression for the kinetic co-energy, the momentum balance relations take the form

\[
d \frac{dt}{dt} \left( m^{(i)} \dot{c}^{(i)} \right) + \frac{\partial V}{\partial c^{(i)}} = f^{(i)} - f'^{(i)} \tag{74}
\]

\[
d \frac{dt}{dt} \left( 4 G^{(i)} J^{(i)} G^{(i)} \dot{e}^{(i)} \right) - k^{(i)} + \frac{\partial V}{\partial e^{(i)}} = 2 G^{(i)} T^{(i)} + \gamma^{e(i)} e^{(i)} \tag{75}
\]

Introducing the momentum variables

\[
p^{(i)} = m^{(i)} \dot{c}^{(i)}, \quad h^{(i)} = J^{(i)} \omega^{(i)} \tag{76}
\]

and premultiplying the angular momentum balance expression (75) by \( \frac{1}{2} G^{(i)} \), which [using equation (7)] removes the last term, yields the final state space formulation

\[
\dot{p}^{(i)} = -g^{(i)} - f'^{(i)} + f^{(i)} \tag{77}
\]

\[
\dot{h}^{(i)} = -\Omega^{(i)} h^{(i)} - \frac{1}{2} G^{(i)} M^{(i)} + T^{(i)} \tag{78}
\]

\[
\dot{c}^{(i)} = m^{-1} p^{(i)} \tag{79}
\]

\[
\dot{e}^{(i)} = \frac{1}{2} G^{(i)} J^{(i)}^{-1} h^{(i)} \tag{80}
\]

Augmented by the evolution equations for \( S^{(i)}, D^{(j)}, d^{(j)}, \) and \( E^{p(j)} \), these are explicit first order equations for the particle-element system. With appropriate initial conditions they may be integrated to obtain solutions for a wide range of hypervelocity impact problems.

**EXAMPLE SIMULATIONS**

This section compares the results of simulations performed using a numerical implementation [27] of the model developed here, to published experimental results for two hypervelocity impact problems. The simulations employed a Mie-Grumisen equation of state and

\[
c_o = 0.001, \quad c_1 = 0.0, \quad k_o = 0.1, \quad \epsilon_f^p = 1.0 \tag{81}
\]
with material properties taken from Steinberg [28].

The first example involves the impact of a uranium alloy rod on a steel plate, 0.64 cm in thickness, in an experiment described by Hertel [29]. The cylindrical projectile is 0.767 cm in diameter, with a length to diameter ratio of ten, and impacts the plate at an obliquity of 73.5 degrees and a velocity of 1.21 kilometers per second. The velocity of the target plate is 0.217 kilometers per second. The simulation employed 712,929 particles, with the projectile composed of spherical particles and the target plate composed of ellipsoidal particles with aspect ratios of 1.5:1.5:1.0. Figure 1 shows an element plot of the initial configuration, while Figure 2 shows a particle plot of the simulation results at 100 microseconds after impact. The simulation results for the rod erosion (27.5 percent) and residual rod velocity (1.10 km/s) show good agreement with the corresponding experimental values (27.6 percent and 1.07 km/s).

The second example involves the impact of a tungsten rod on a steel plate, 0.95 cm in thickness, in an experiment described by Yatteau et al. [30]. The cylindrical projectile is 0.475 cm in diameter, with a length to diameter ratio of twenty, and impacts the plate at an obliquity of 75 degrees and a velocity of 1.83 kilometers per second. The target plate is stationary. The simulation employed 671,176 particles, with the projectile composed of spherical particles and the target plate composed of ellipsoidal particles with aspect ratios of 1.5:1.5:1.0. Figure 3 shows an element plot of the initial configuration, while Figure 4 shows a particle plot of the simulation results at 150 microseconds after impact. The simulation results for the rod erosion (37.9 percent) and residual rod velocity (1.60 km/s) show good agreement with the corresponding experimental values (40 percent and 1.78 km/s).

In the preceding problems the vast majority of the particles are associated with the target plates, so that the use of ellipsoidal particles reduces the total particle count by a factor of approximately 2.25, equal to the product of the three indicated aspect ratios. It should be noted that the computational advantages of a reduced particle count can in part be offset by an increase in the average number of nominal neighbors per particle. This increase is
due to the fact that the neighbor search for each particle must consider a spatial volume proportional to the cube of the major ellipsoidal axis, despite the fact that a closer approach without contact can occur when particle minor axes are appropriately aligned. Experience to date indicates that the use of ellipsoidal particles leads to a reduction in memory requirements which is proportional to the reduction in particle count, but to a cpu time requirement similar to that measured in corresponding simulations with spherical particles.

CONCLUSION

The present paper has developed an ellipsoidal particle-finite element method for hypervelocity impact problems, and demonstrated its use in the simulation of three dimensional impact experiments. The hybrid particle-element kinematic scheme allows for the use of true Lagrangian strength models, while incorporating a completely general description of contact-impact effects. Tensile instability, numerical fracture, and angular momentum balance problems, often associated with particle formulations, are avoided. Material remaps, mass or energy discard, slideline algorithms, and adaptive mesh refinement, often associated with continuum models, are also avoided. Derivation of the model is based on a thermomechanical form of Lagrange’s equations, with entropy variables as generalized coordinates. The use of an energy based modeling approach facilitates the systematic integration of diverse particle based and element based interpolations. Work to date suggests that the method provides a numerically robust and accurate approach to the simulation of hypervelocity impact problems.

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Figure 2. Uranium alloy long rod impact on a steel plate, particle plot at 100 microseconds after impact

Figure 3. Tungsten long rod impact on a steel plate, element plot of the initial configuration

Figure 4. Tungsten long rod impact on a steel plate, particle plot at 150 microseconds after impact
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