

# Stirling Analysis Comparison of Commercial VS. High-Order Methods



Rodger W. Dyson, Scott D. Wilson

Roy C. Tew, and Rikako Demko

Thermal Energy Conversion Branch

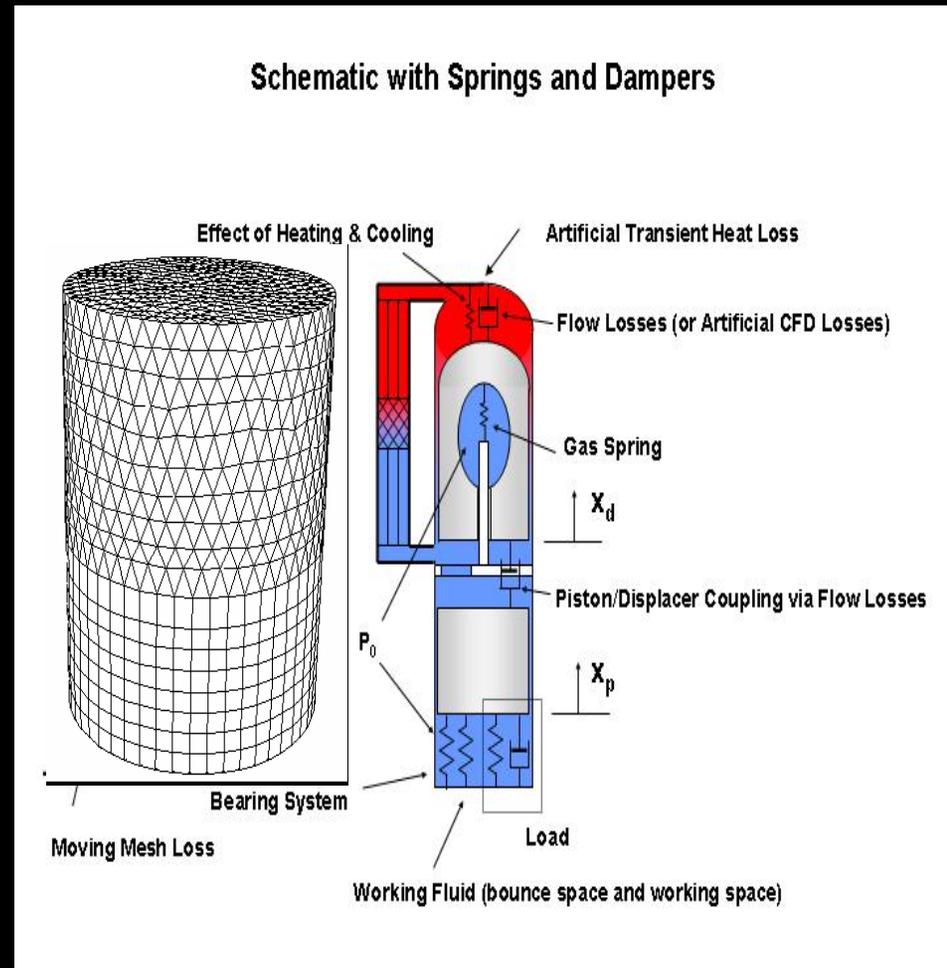
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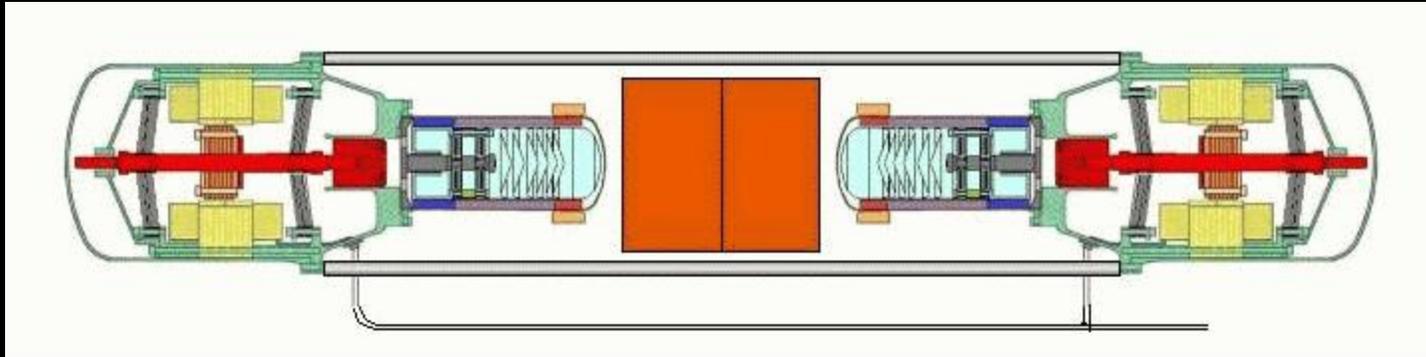
# Stirling Simulation Numerical Error

- Remeshing interpolation
- Layering interpolation
- Diffusive time advance
- Low grid quality/skewness
- Sliding interface interpolation
- Artificial entropy
- Turbulence transition



# Dual Opposed Convertors

- High Efficiency – Low Mass Space Power

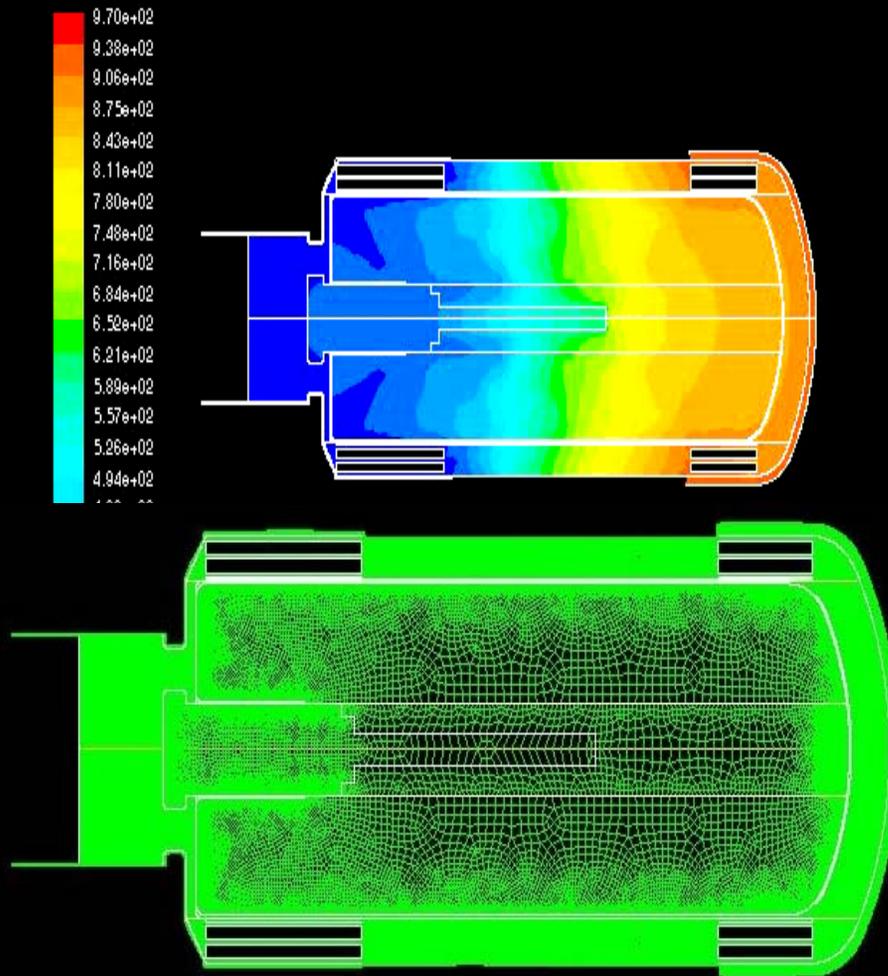


- Free Piston Geometry is Essentially Smooth

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# Whole Engine Simulation

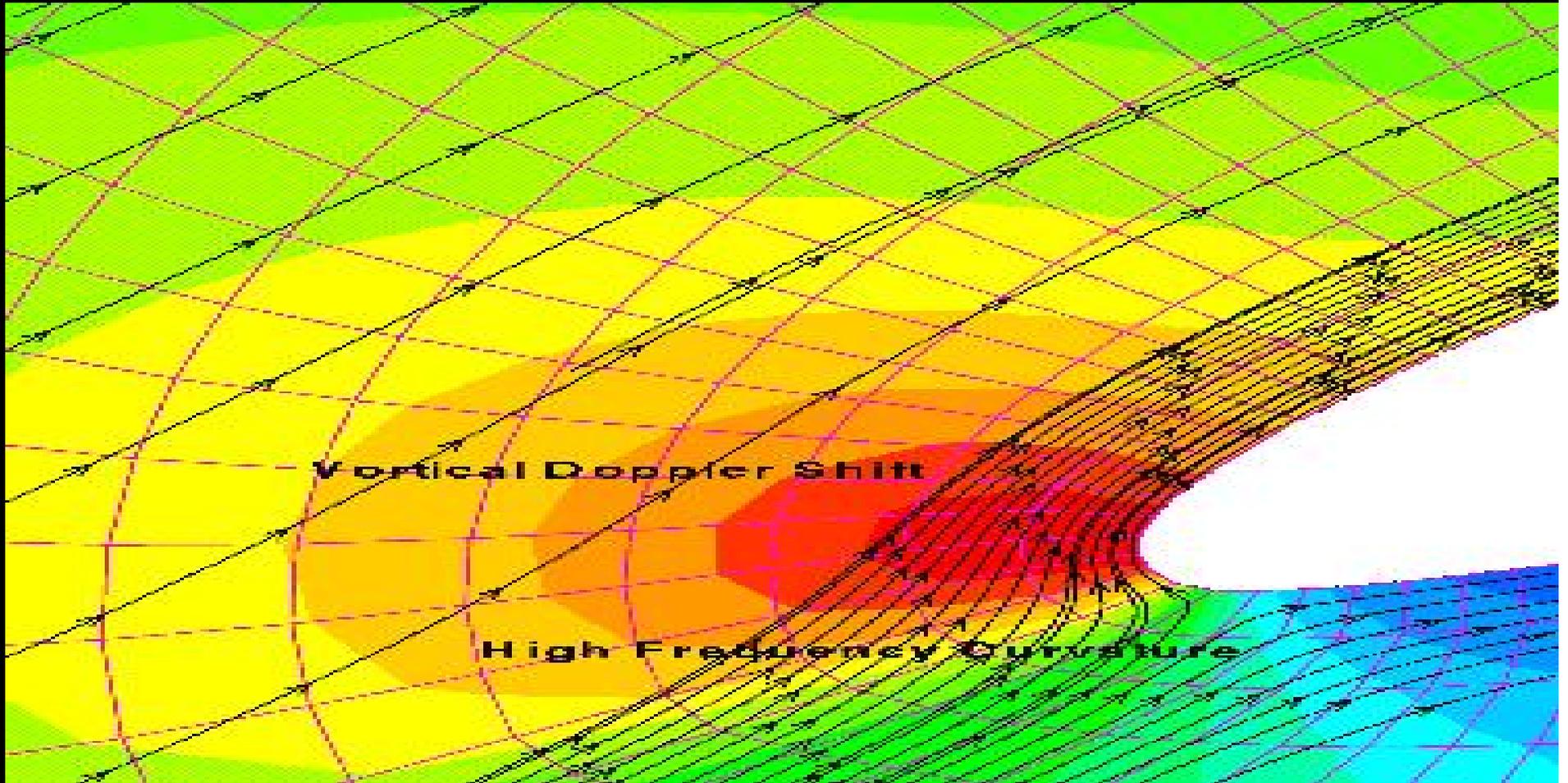


- Bounded by Walls – No need for nonreflecting B.C.
- Kolmogorov scales fairly large
- Steep thermal gradients
- No shocks/subsonic/transitioning
- High-order friendly

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# Curvilinear Features



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# Stability Analysis

- Courant (CFL) number,  
 $r = c \Delta t / \Delta x$
- Von Neumann number,  
 $v = \mu \Delta t / \Delta x^2$
- Linear Viscous Burger's  
Equation

$$\frac{\partial u}{\partial t} + c \frac{\partial u}{\partial x} = \mu \frac{\partial^2 u}{\partial x^2}$$



## Compact Scheme 6<sup>th</sup> Order in Space

$$\alpha \left( \frac{\partial u}{\partial x} \right)_{i-1} + \left( \frac{\partial u}{\partial x} \right)_i + \alpha \left( \frac{\partial u}{\partial x} \right)_{i+1} = a \frac{u_{i+1} - u_{i-1}}{2\Delta x} + b \frac{u_{i+2} - u_{i-2}}{4\Delta x}$$

$$\alpha = 1/3, a = 14/9, b = 1/9$$

$$\alpha \left( \frac{\partial^2 u}{\partial x^2} \right)_{i-1} + \left( \frac{\partial^2 u}{\partial x^2} \right)_i + \alpha \left( \frac{\partial^2 u}{\partial x^2} \right)_{i+1} = a \frac{u_{i+1} - 2u_i + u_{i-1}}{\Delta x^2} + b \frac{u_{i+2} - 2u_i + u_{i-2}}{4\Delta x^2}$$

$$\alpha = 2/11, a = 12/11, b = 3/11$$

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# Runge-Kutta 4<sup>th</sup> Order

$$R(u) = -cu_x + \mu u_{xx}$$

$$u^{(1)} = u^n + \frac{\Delta t}{2} R^n$$

$$u^{(2)} = u^n + \frac{\Delta t}{2} R^1$$

$$u^{(3)} = u^n + \Delta t R^2$$

$$u^{(n+1)} = u^n + \frac{\Delta t}{6} (R^n + 2R^{(1)} + 2R^{(2)} + R^{(3)})$$

$$R^{(1)} = R(u^{(1)}), R^{(2)} = R(u^{(2)}), R^{(3)} = R(u^{(3)})$$



## Current Practice

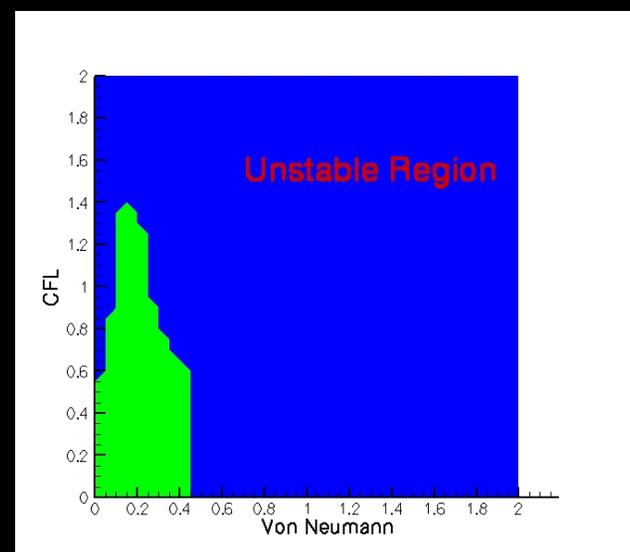
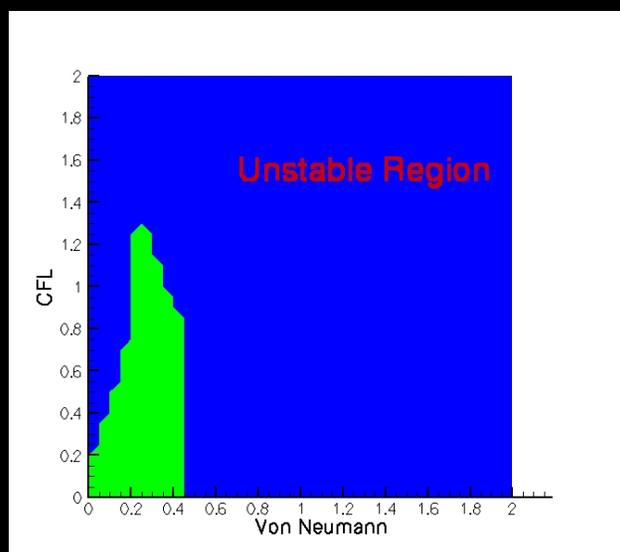
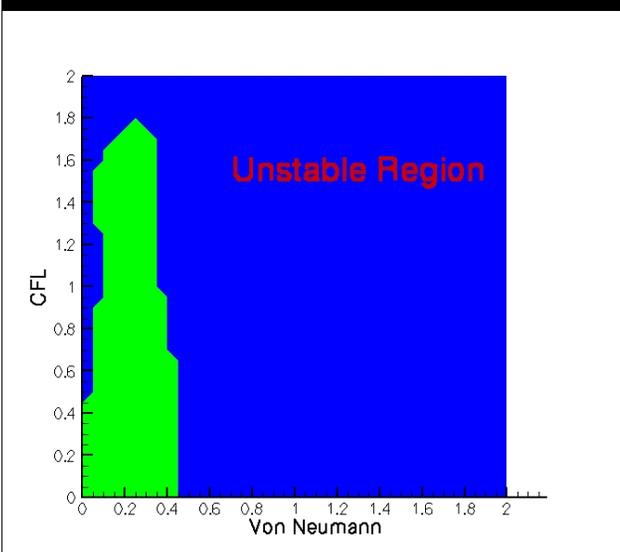
- Implicit 1<sup>st</sup> or 2<sup>nd</sup> order in time commercially
- 1<sup>st</sup> or 2<sup>nd</sup> order in space implicit
- Explicit/implicit 4<sup>th</sup> order in time academically
- Implicit 6<sup>th</sup> order compact scheme in space

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# Compact Scheme Stability Range

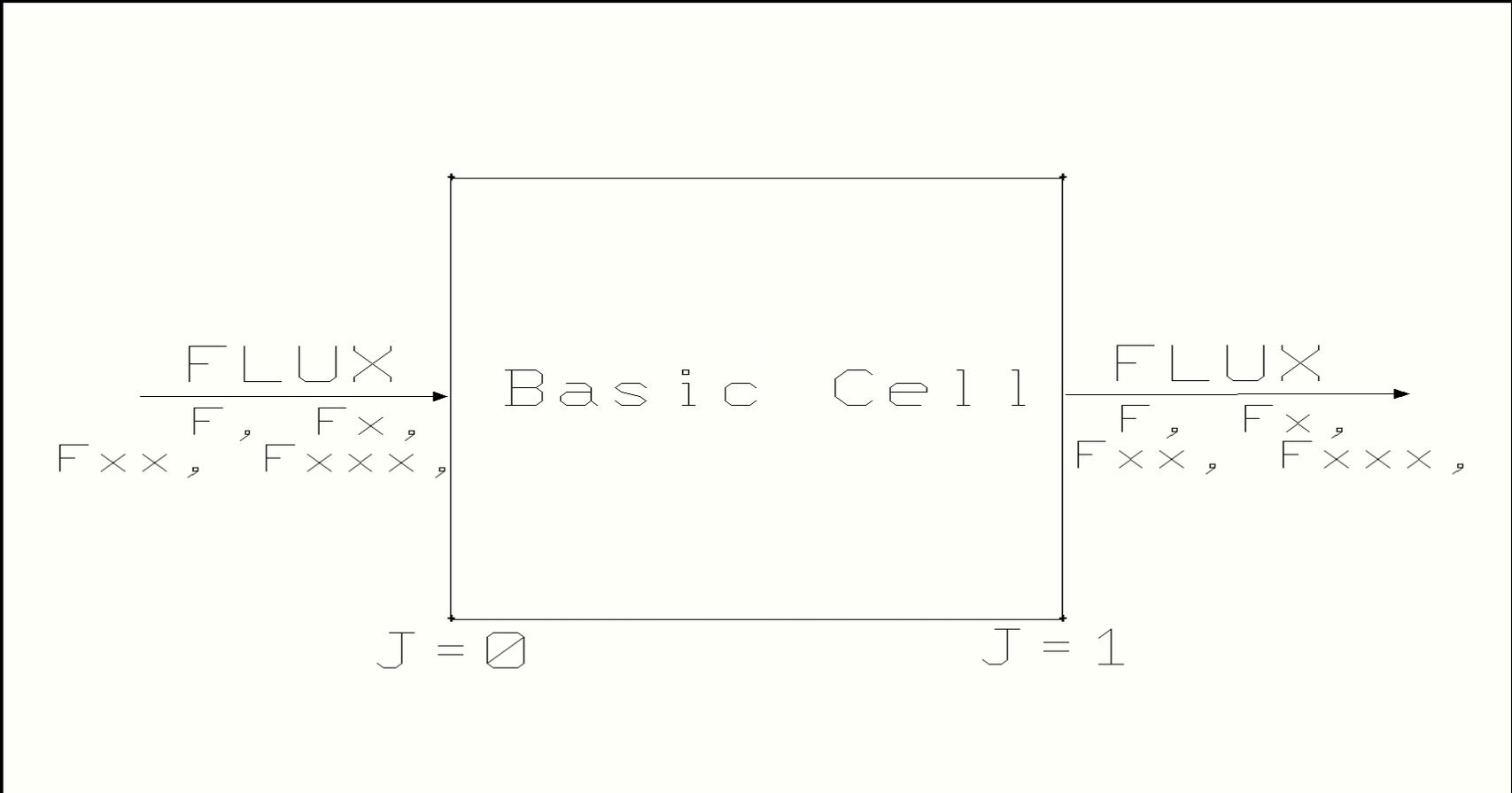
- Domain size affects stability since implicit



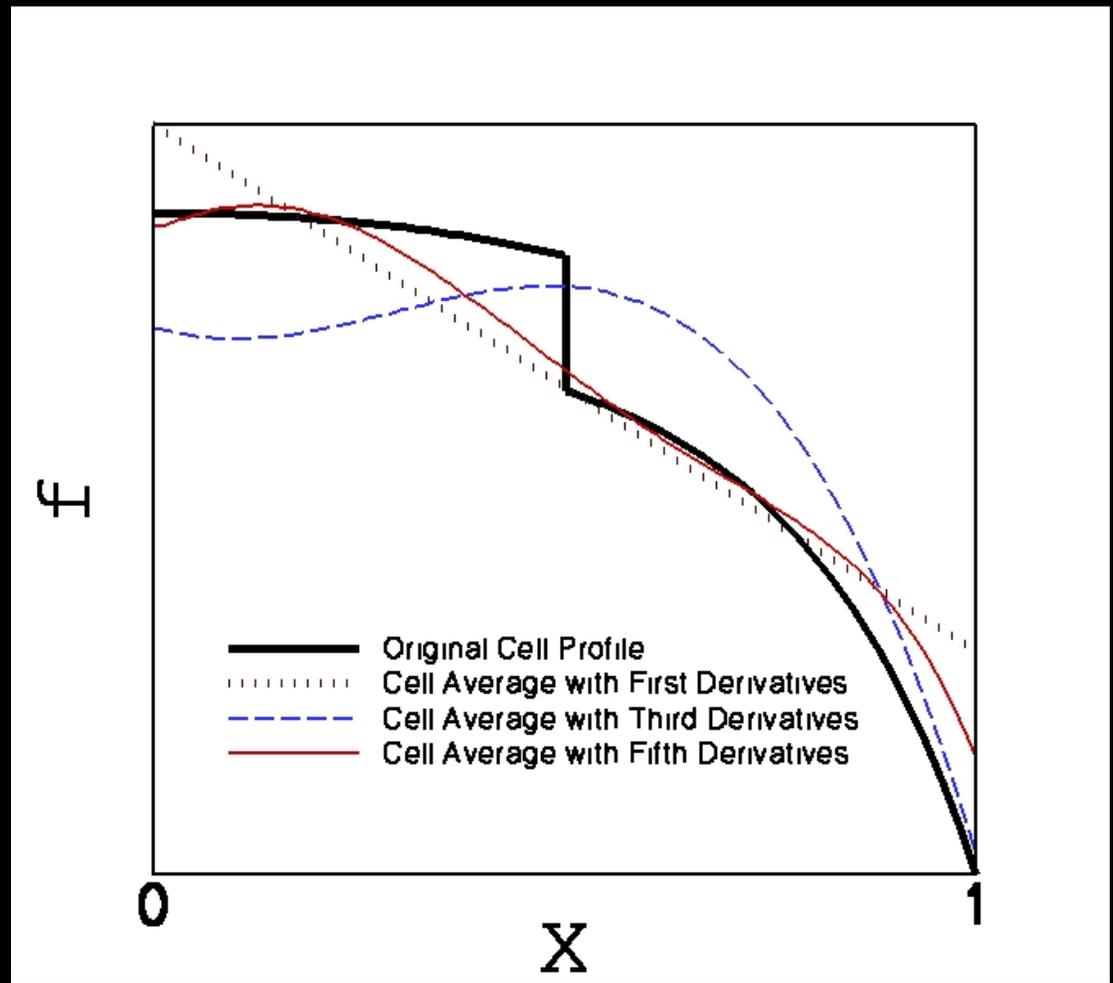
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# Basic UHF Technique



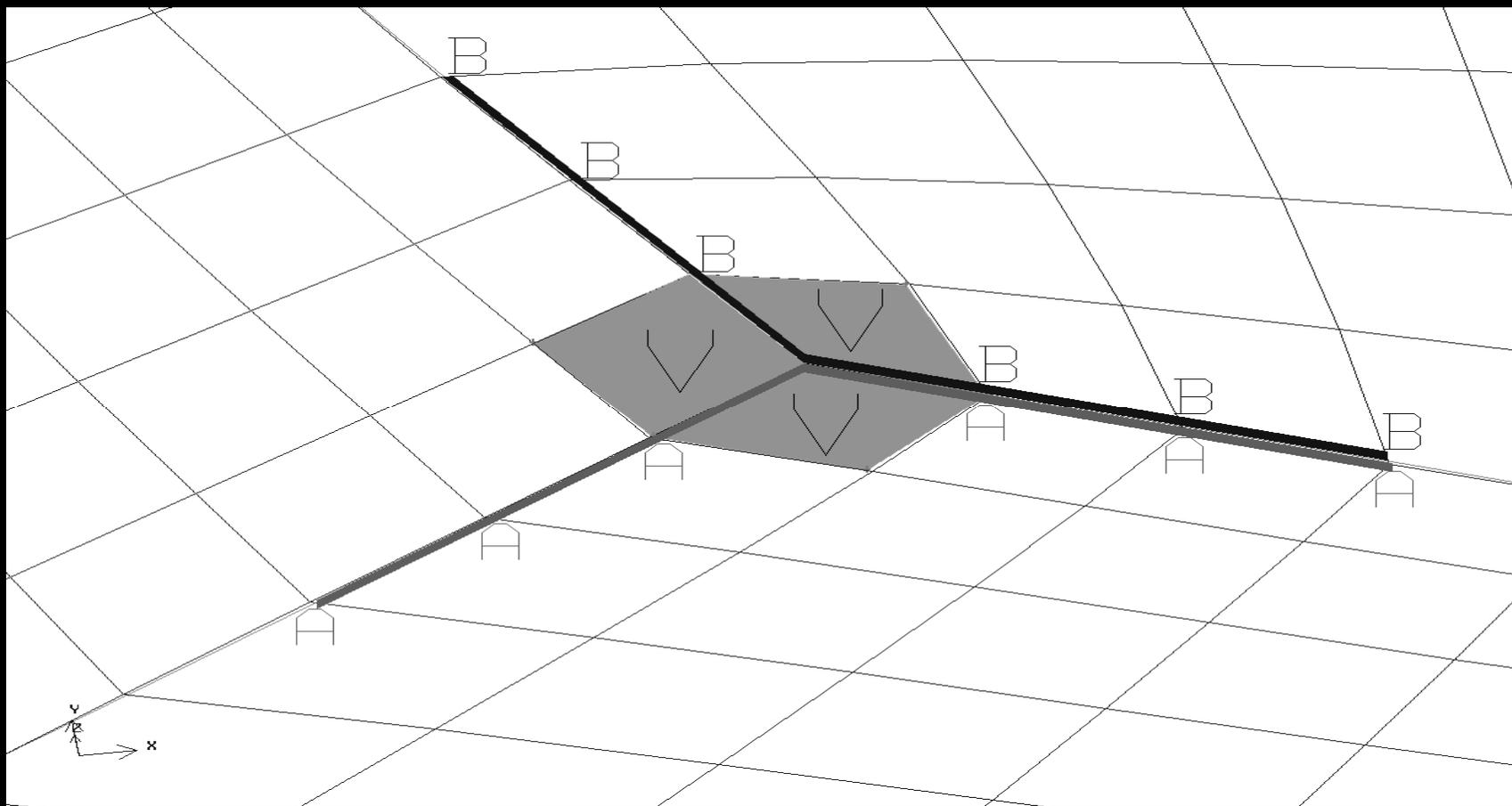
# Derivatives of Cell Averages



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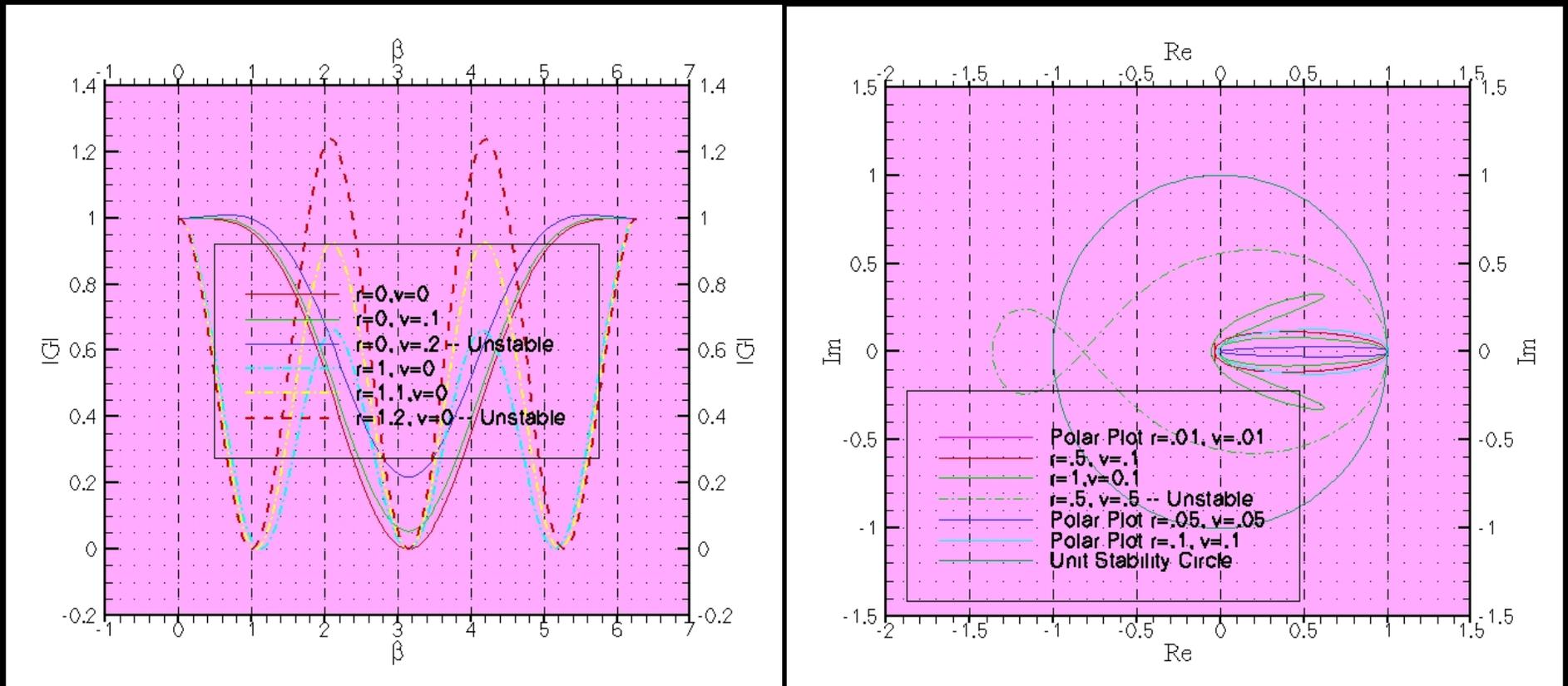
# Grid Singularity Resolution



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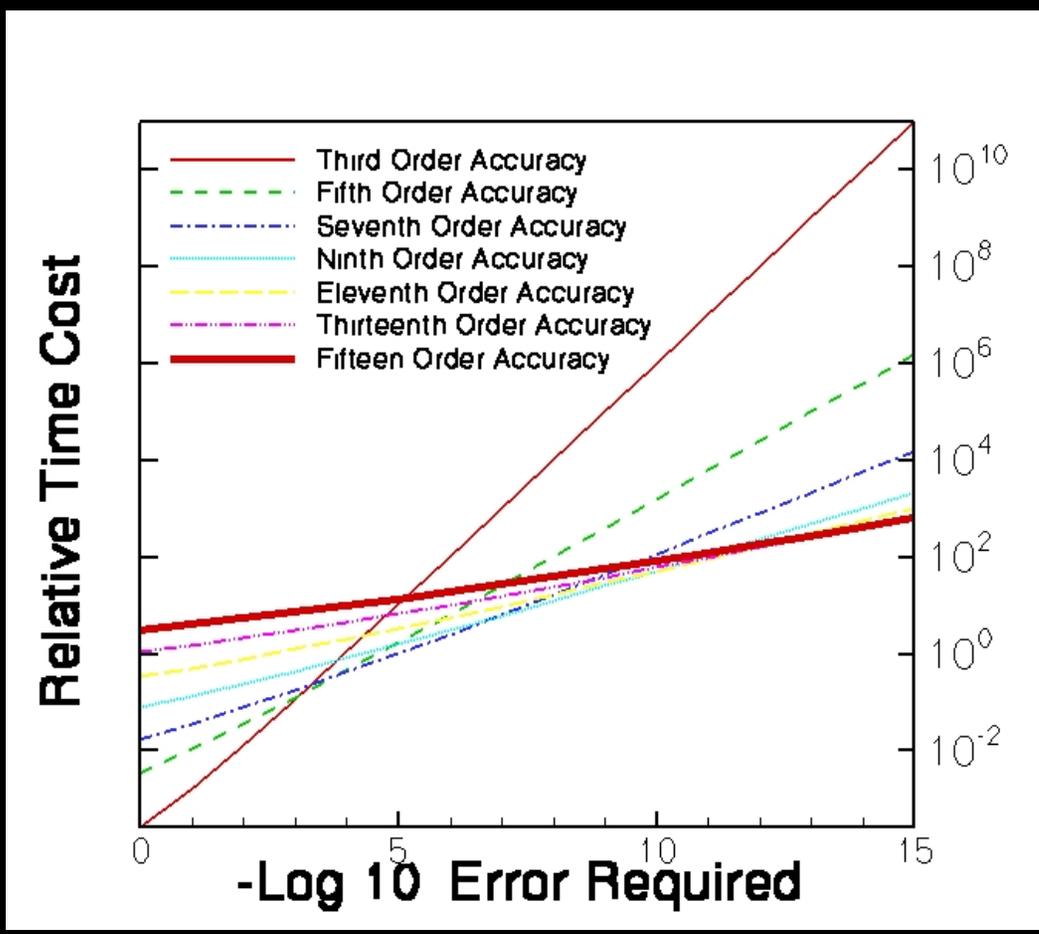
# C4o0 Linear Viscous Burger's Equation



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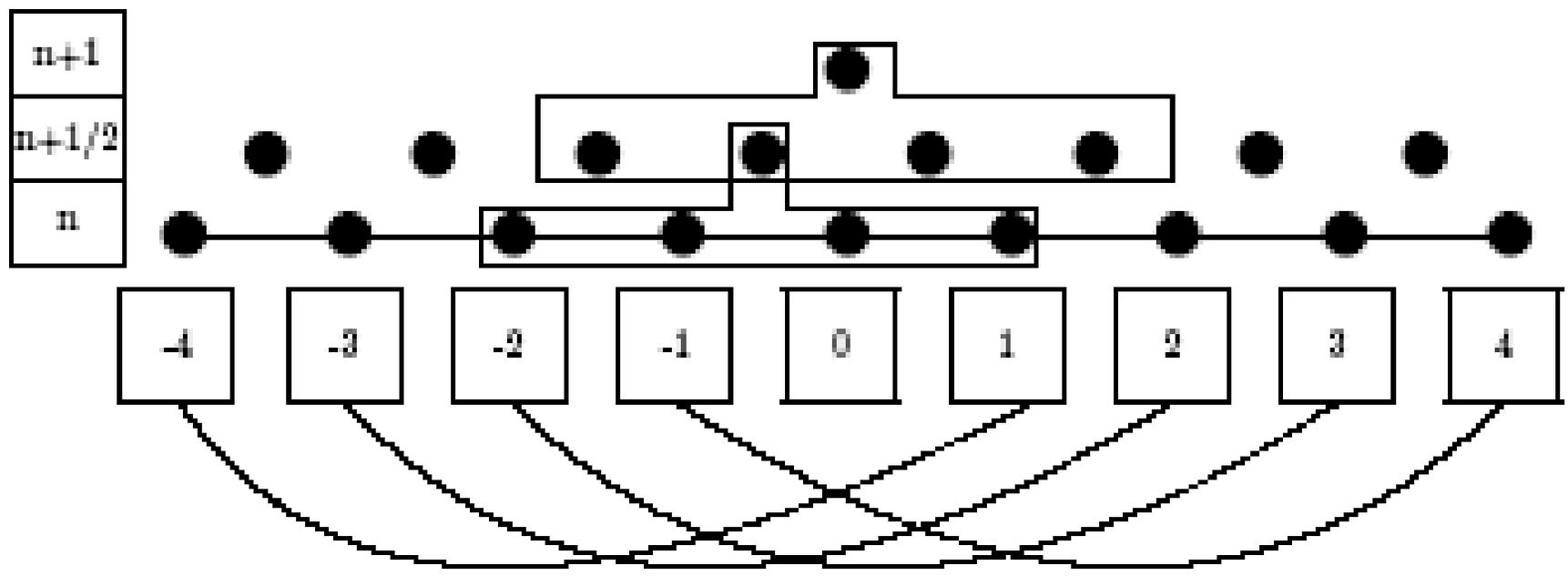
# Efficiency Improves with Accuracy



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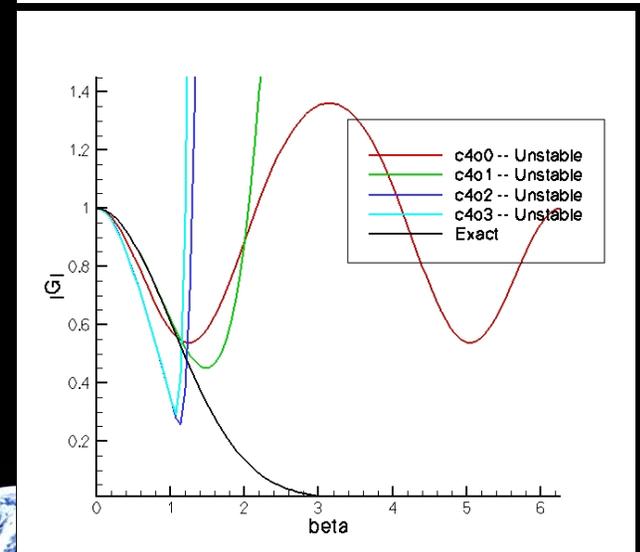
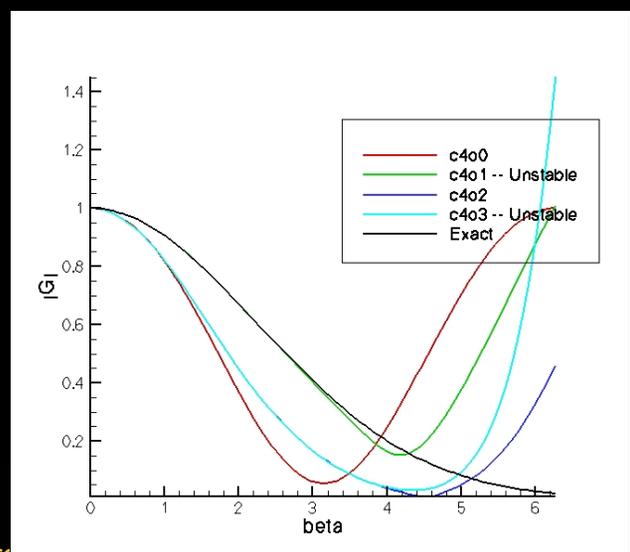
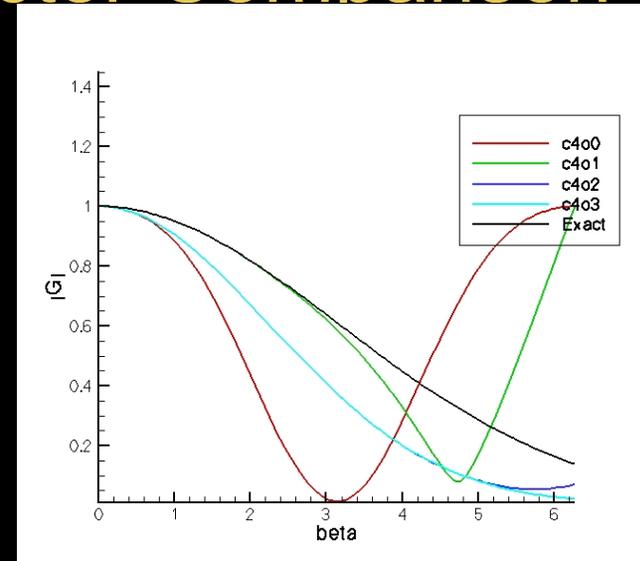
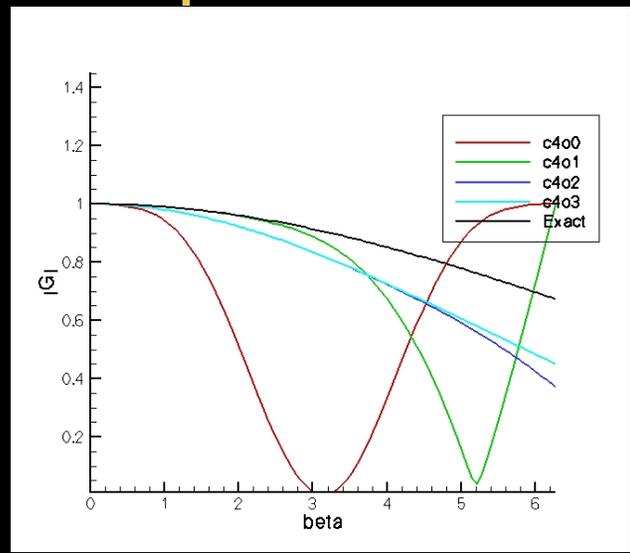
# Computational Domain Schematic



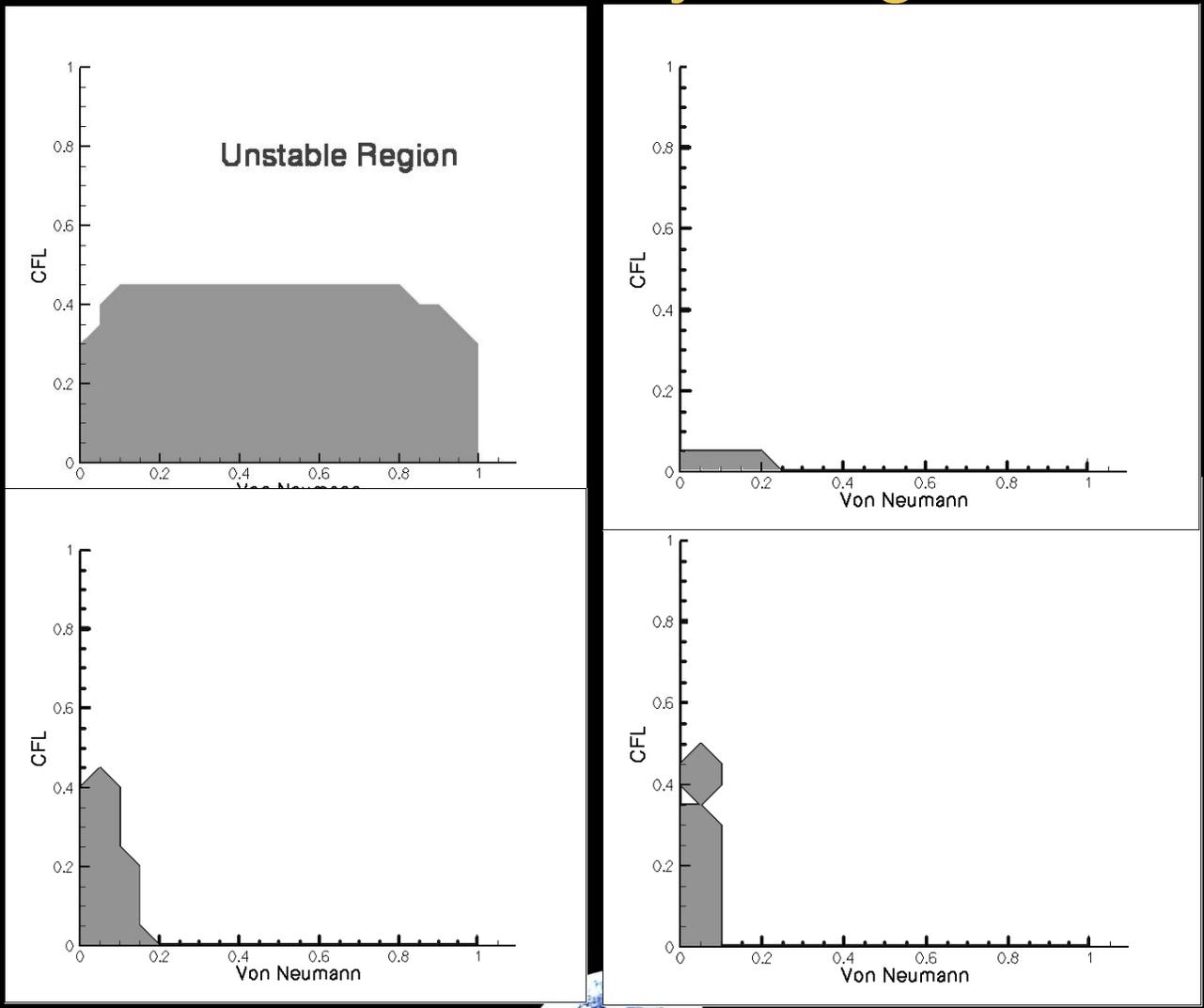
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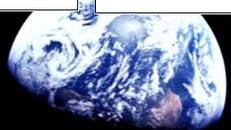
# Amplification Factor Comparison



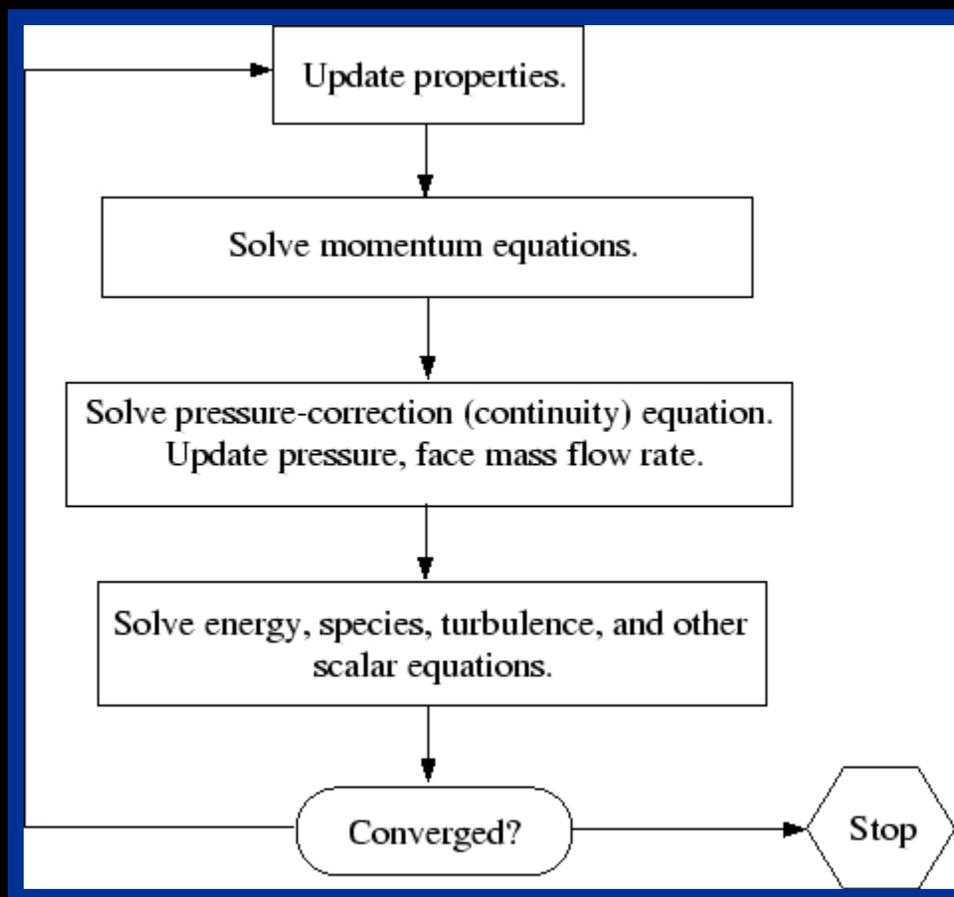
# UHF Stability Range



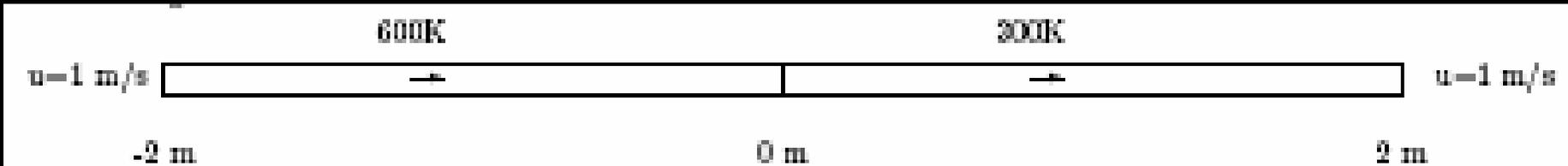
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# Overview of Segregated Solution



# Heat Transfer Test



$$\frac{\partial E_t}{\partial t} + \frac{\partial}{\partial x} \left( (\rho C_v T + p)u - \frac{4}{3} \mu u_x + q_x \right) = 0$$

Reduces to:

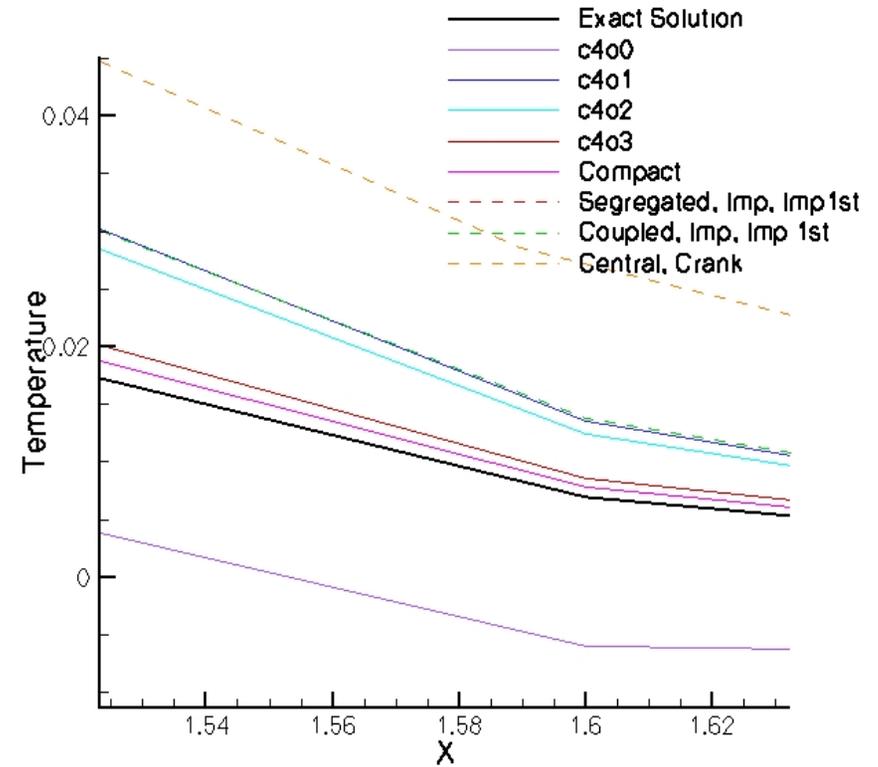
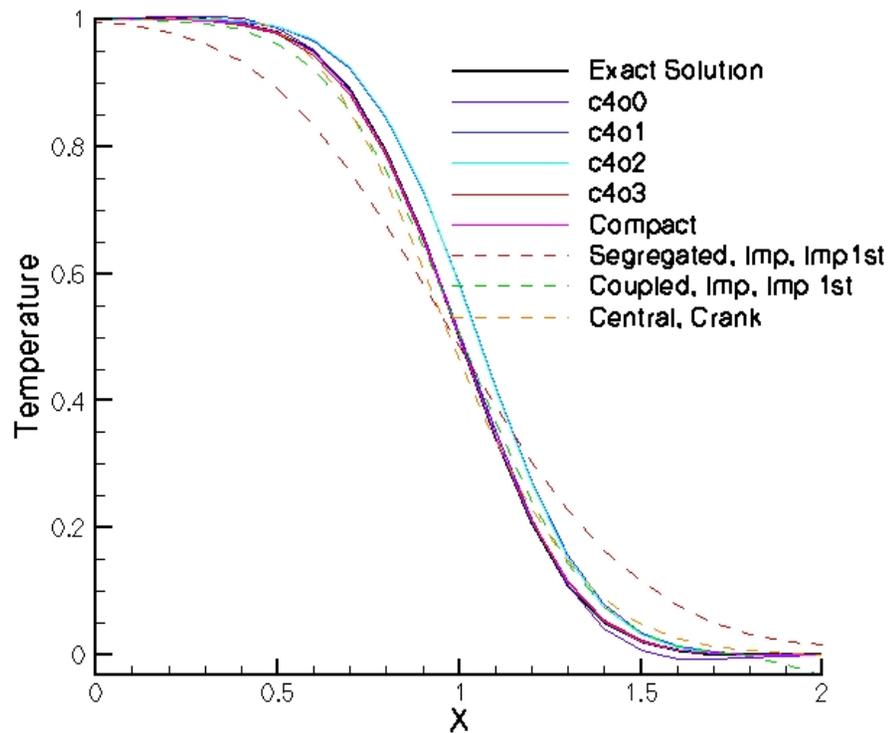
$$\frac{\partial T}{\partial t} + u \frac{\partial T}{\partial x} = \alpha \frac{\partial^2 T}{\partial x^2},$$

$$\alpha = \frac{k}{\rho C_p}$$

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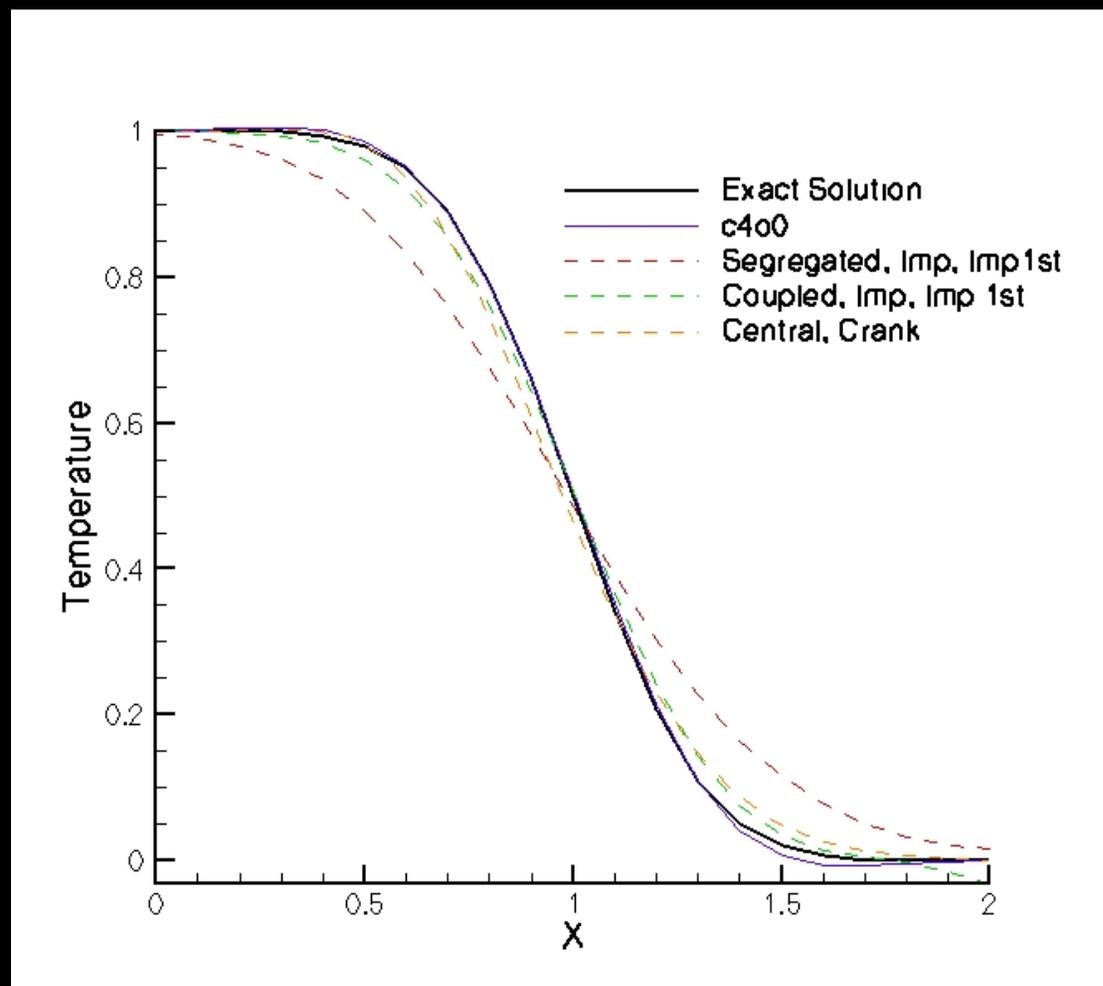
# Comparison of Commercial & Advanced



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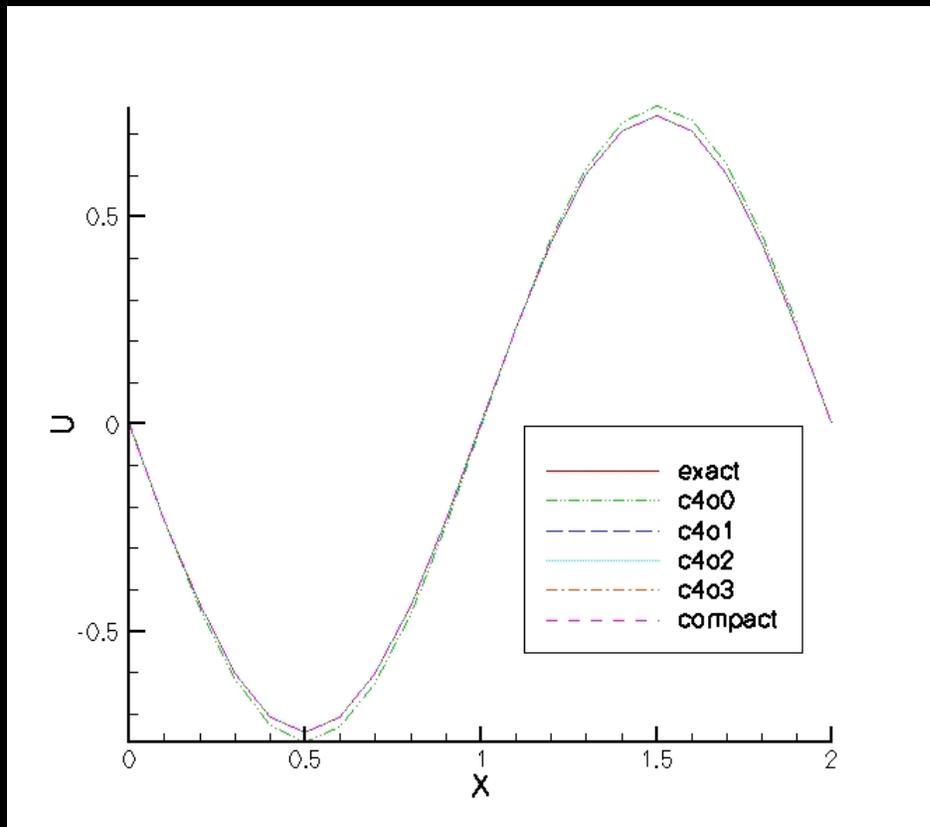
# Commercial Comparison



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# Turbulence Transition Efficiency

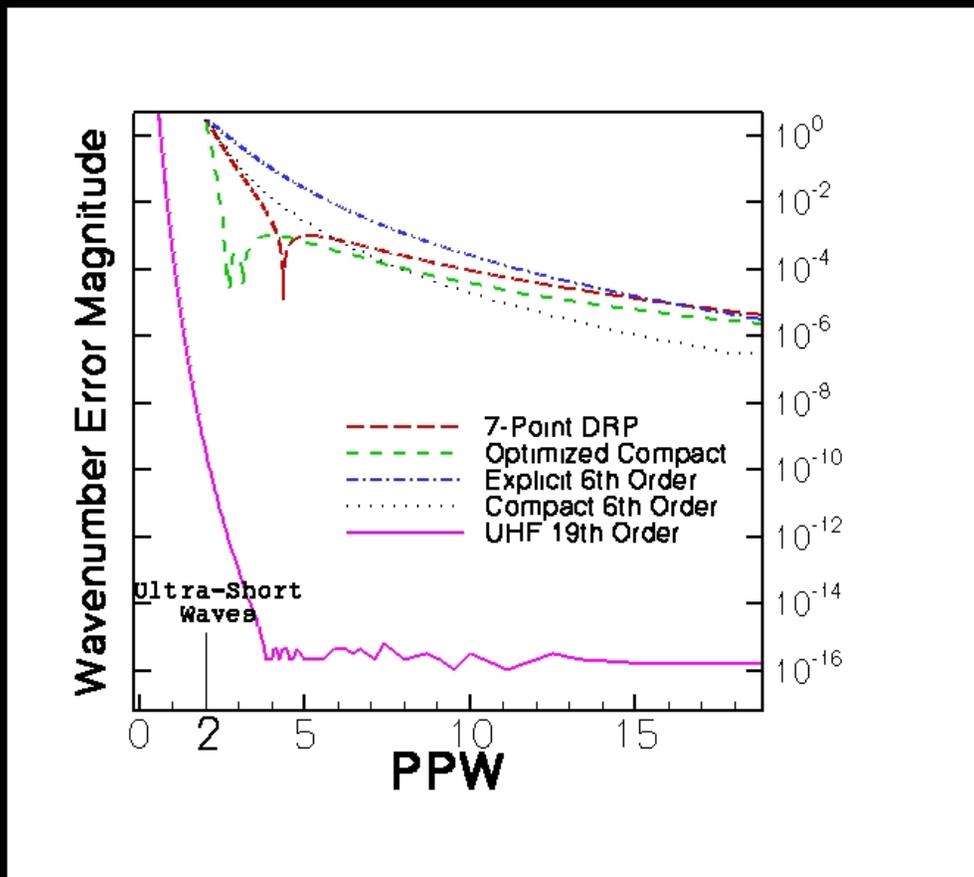


| Method  | Spacing | Error                    |
|---------|---------|--------------------------|
| c4o0    | .1      | $2.54299 \cdot 10^{-2}$  |
| c4o0    | .2      | $4.27563 \cdot 10^{-2}$  |
| c4o0    | .4      | $4.68577 \cdot 10^{-2}$  |
| c4o1    | .1      | $3.11163 \cdot 10^{-6}$  |
| c4o1    | .2      | $2.96551 \cdot 10^{-5}$  |
| c4o1    | .4      | $8.4702 \cdot 10^{-4}$   |
| c4o2    | .1      | $1.0178 \cdot 10^{-10}$  |
| c4o2    | .2      | $3.1935 \cdot 10^{-9}$   |
| c4o2    | .4      | $6.41079 \cdot 10^{-8}$  |
| c4o3    | .1      | $3.44169 \cdot 10^{-13}$ |
| c4o3    | .2      | $2.27818 \cdot 10^{-12}$ |
| c4o3    | .4      | $3.12925 \cdot 10^{-11}$ |
| compact | .1      | $1.0993 \cdot 10^{-6}$   |
| compact | .2      | $7.10999 \cdot 10^{-5}$  |
| compact | .4      | $5.31245 \cdot 10^{-3}$  |

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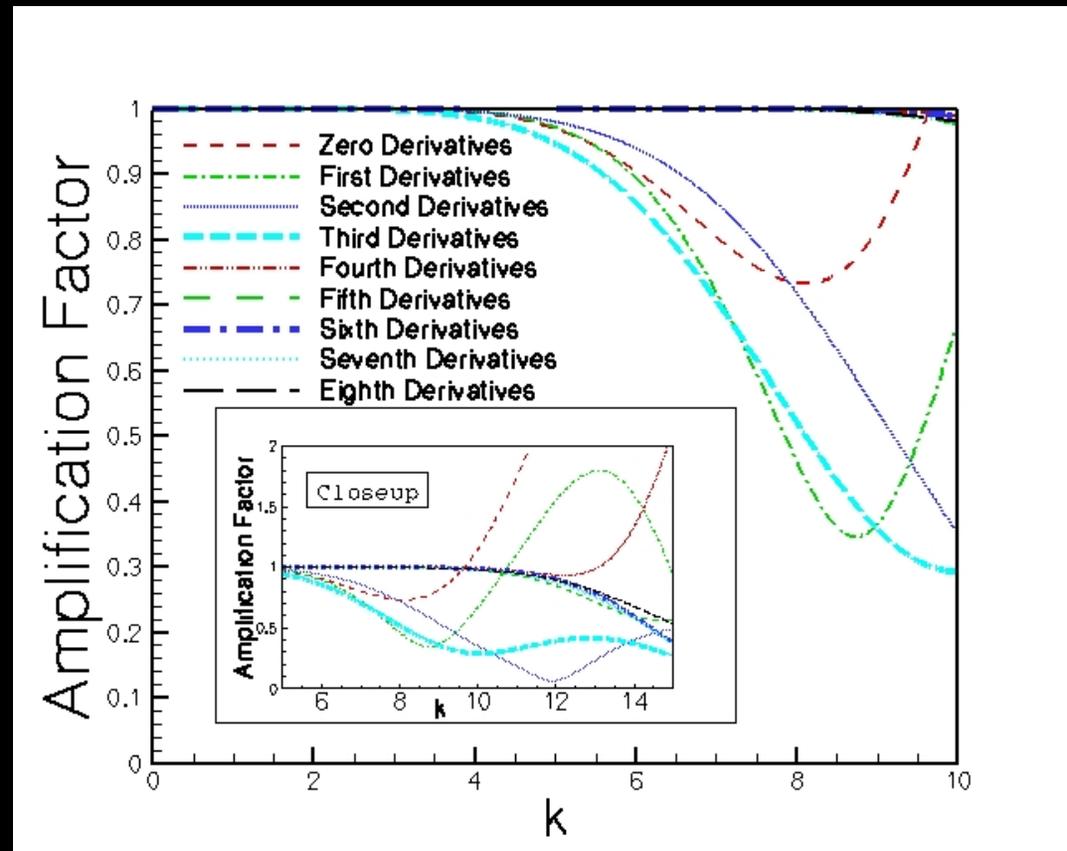
# Points per Kolmogorov Wavelength



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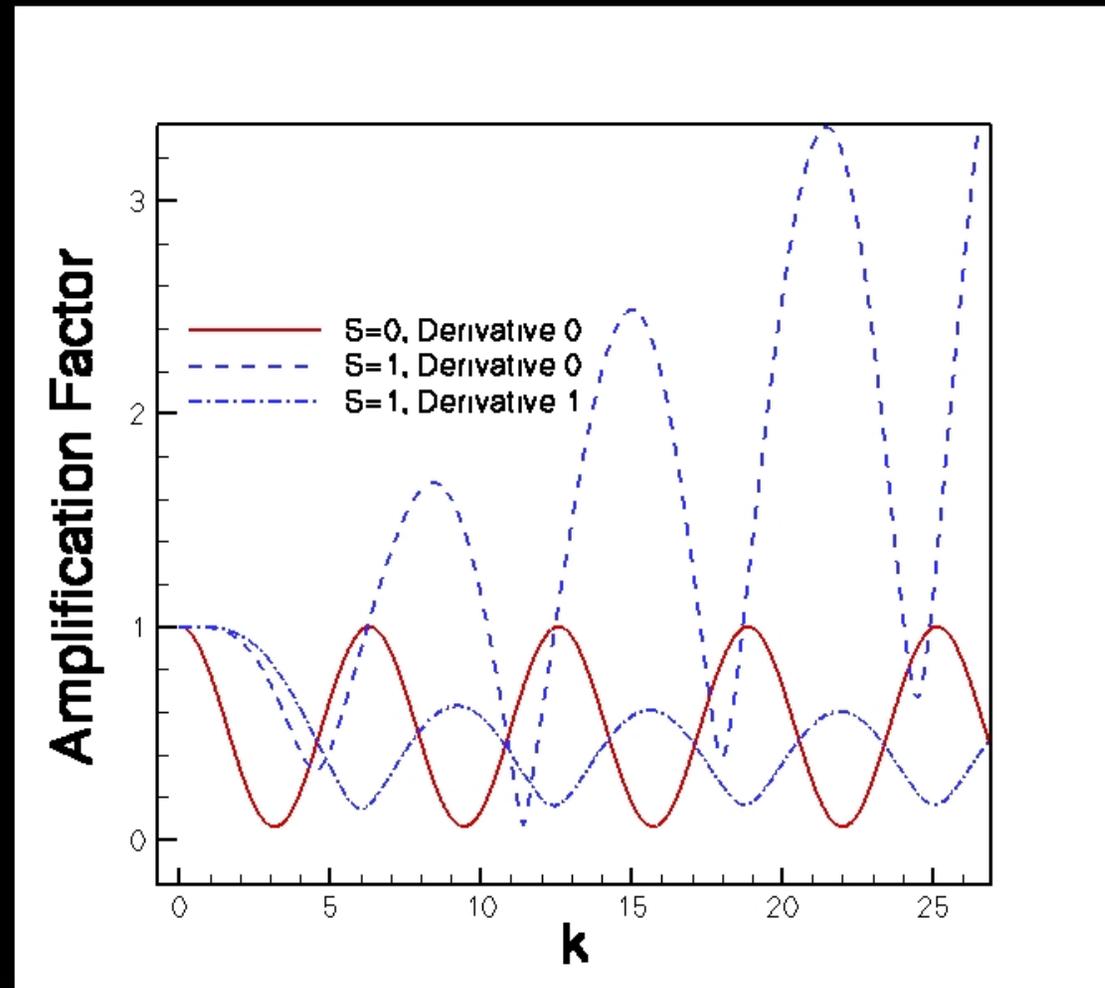
# Wave Equation Amplification



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# Aliased Frequency Amplification



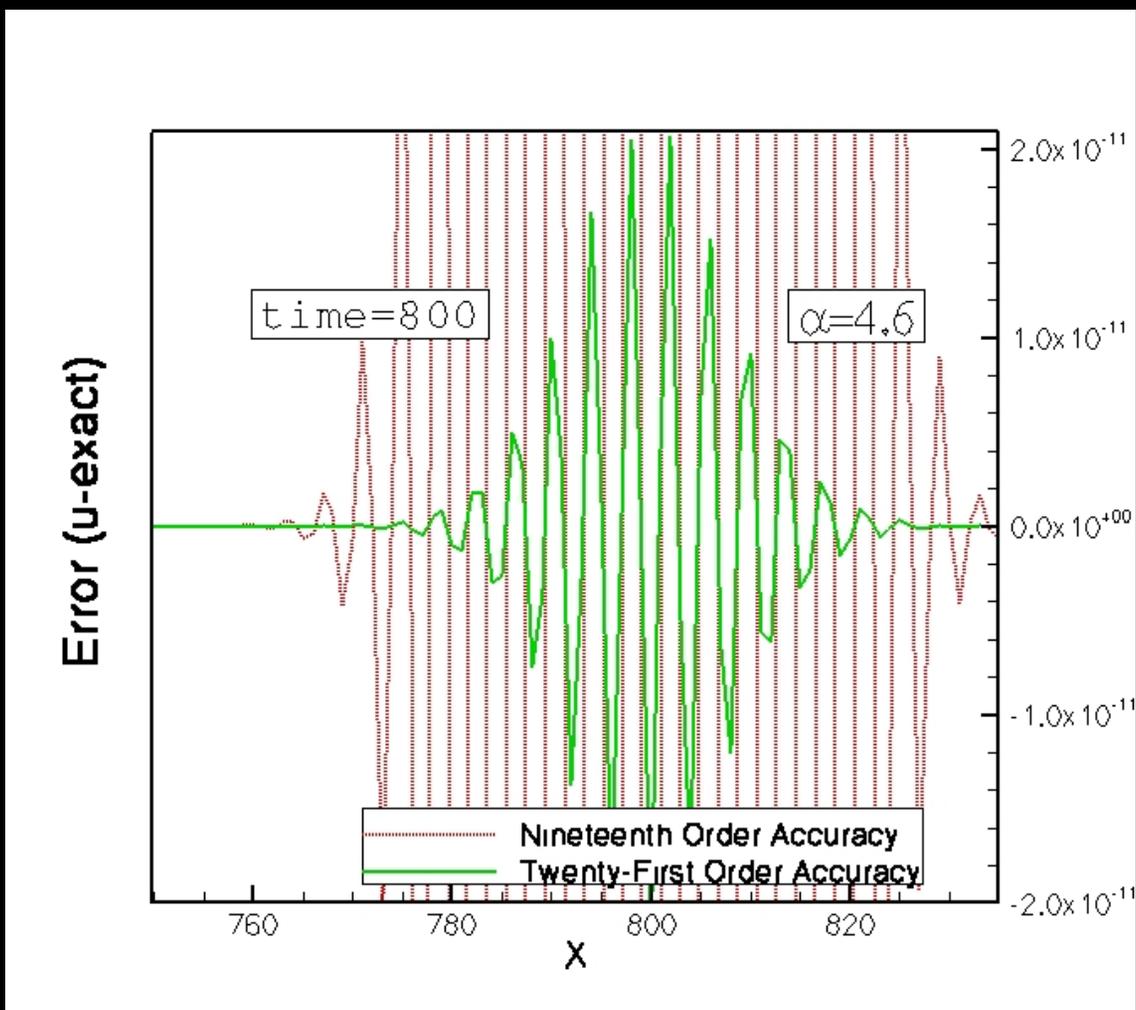
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## Conclusions

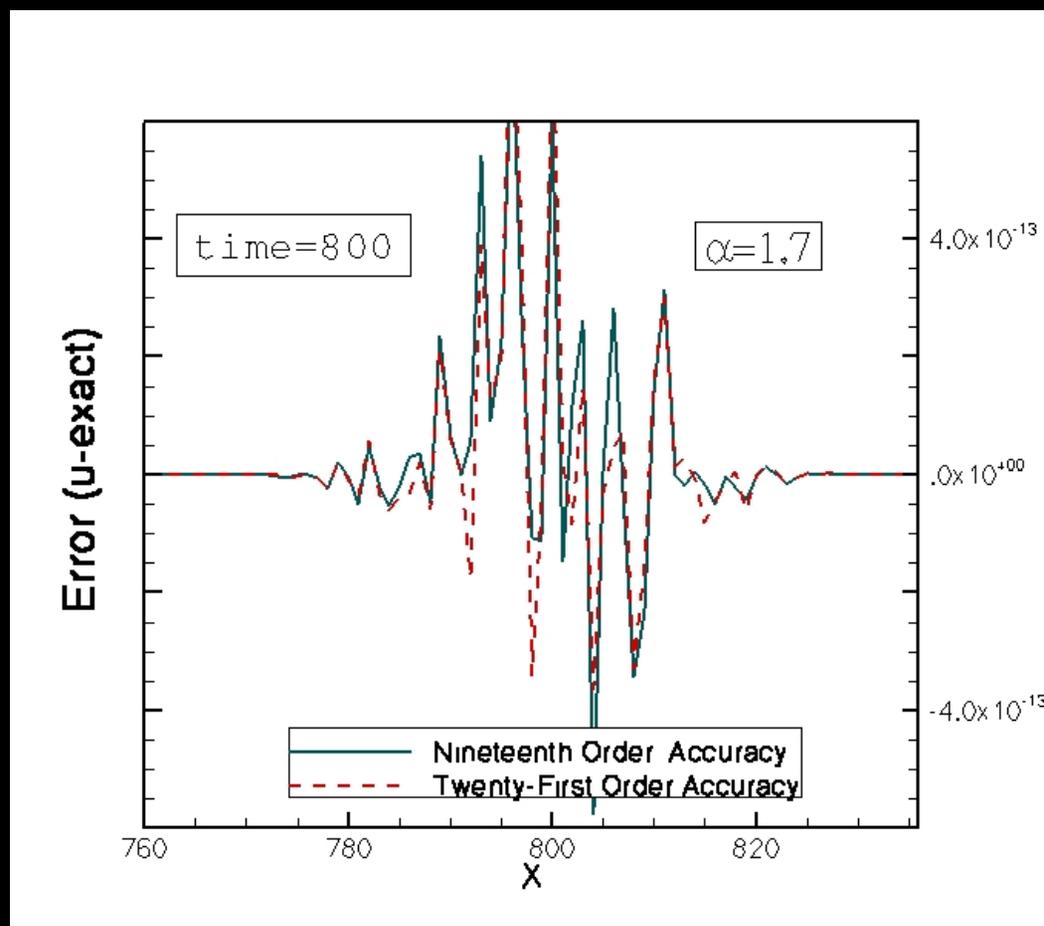
- Low Reynold's number, wall bounded flow allows economical use of large eddy simulation for turbulent transition modeling
- UHF and Compact comparable at conjugate heat transfer
- UHF much better for turbulence modeling
- Modern methods much more efficient than those currently available commercially





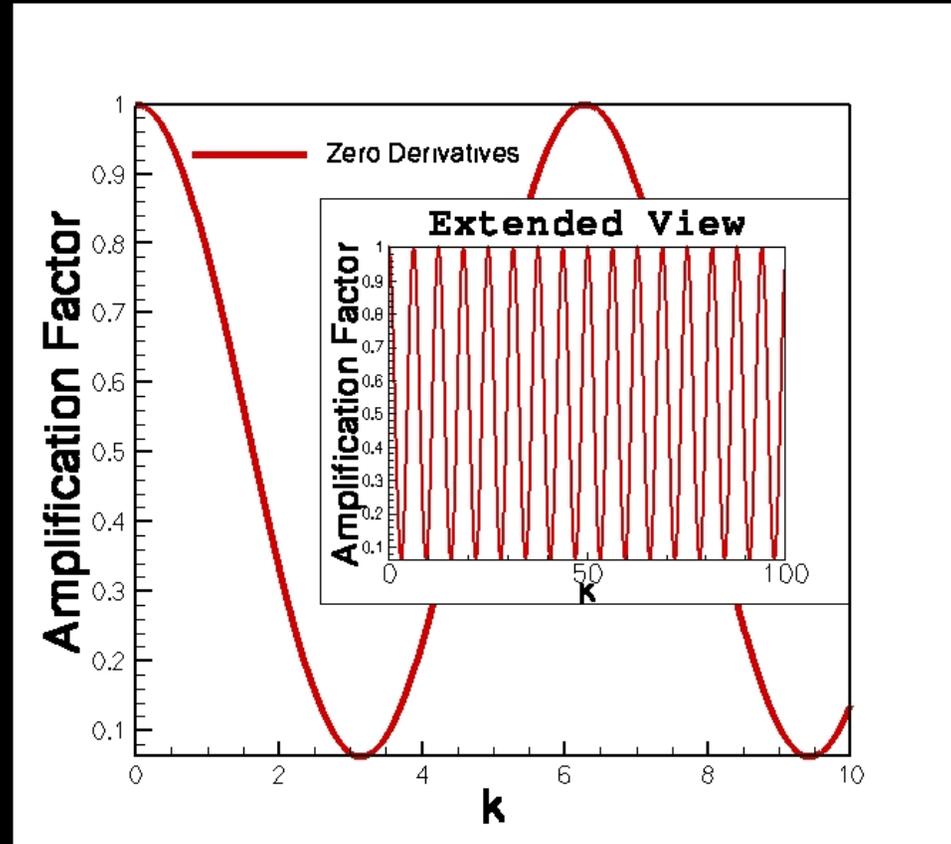
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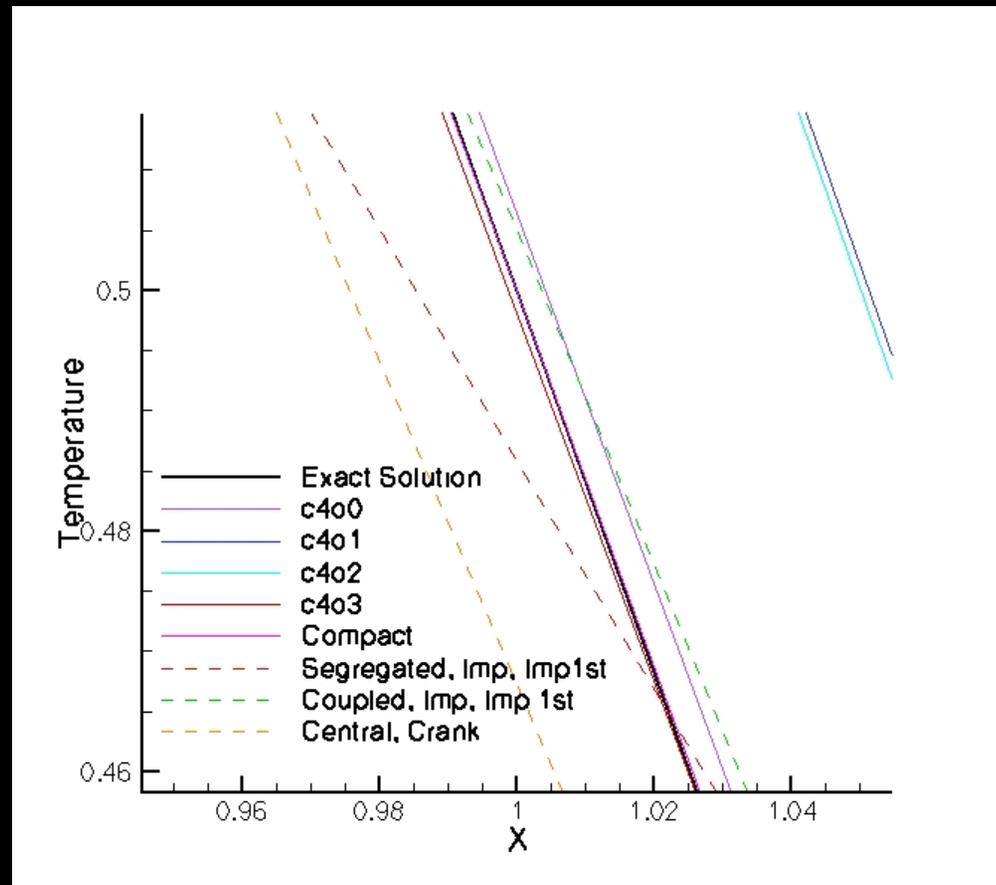
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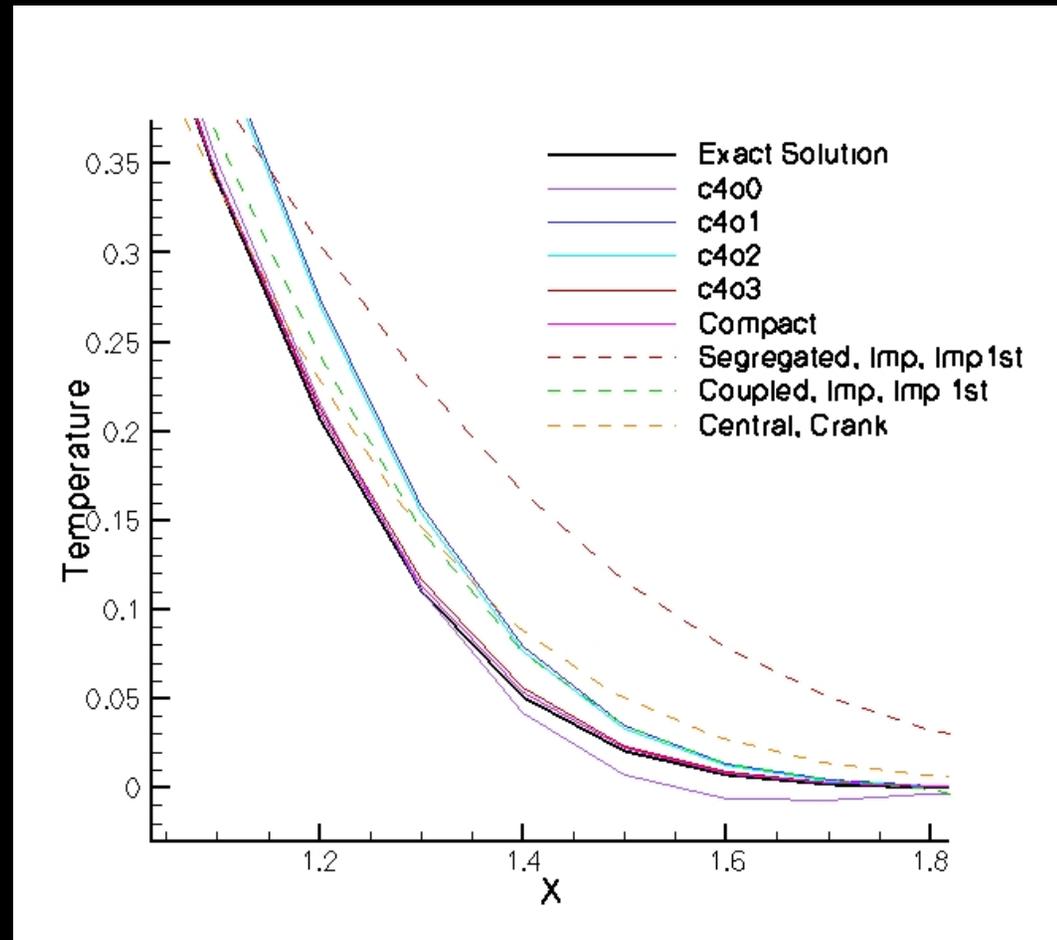
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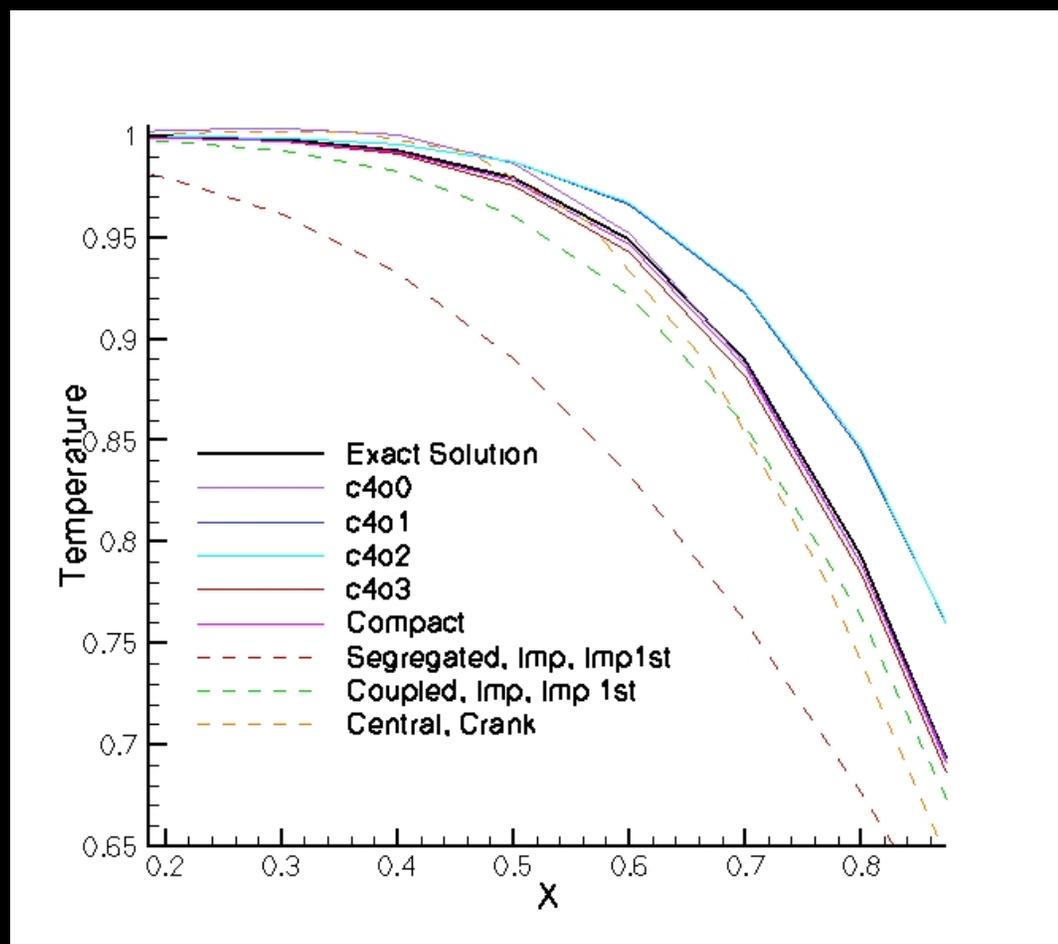
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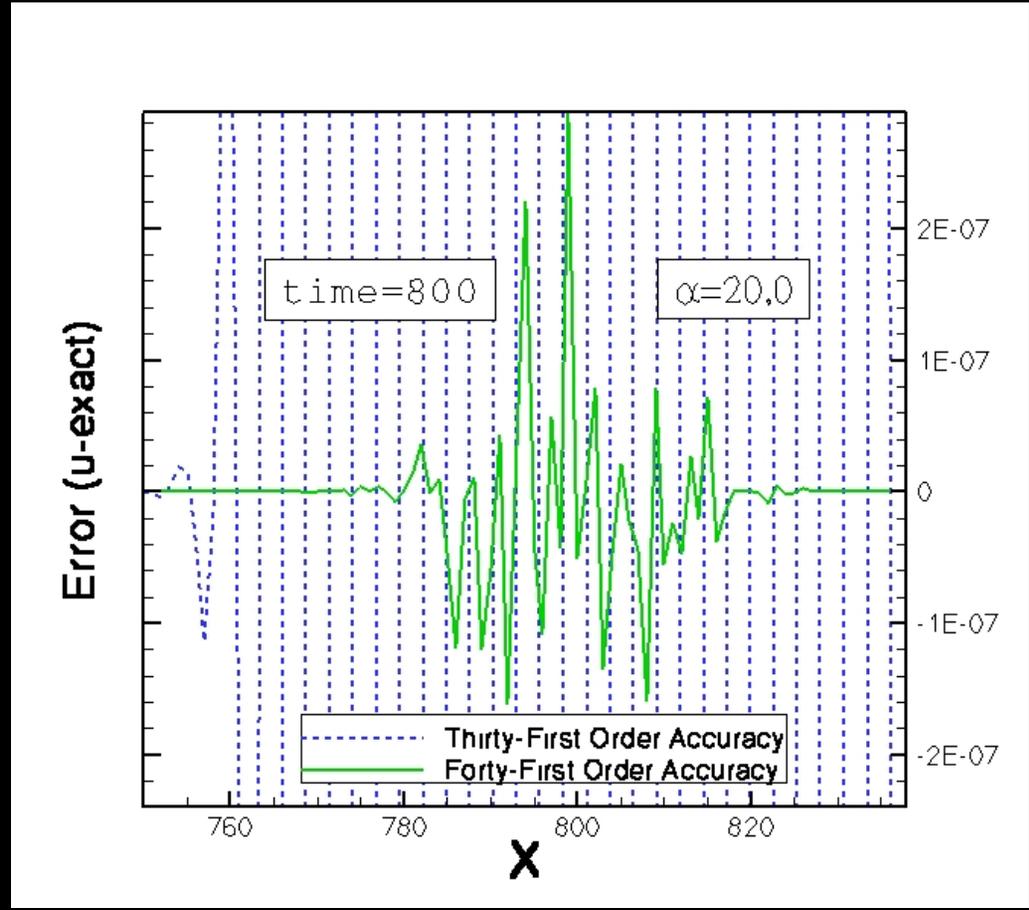
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