Implementation of Speed Variation in the Structural Dynamic Assessment of Turbomachinery Flow-Path Components

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Introduction and Motivation

- Structural \( S_{ult} \) & HCF assessment critical for turbomachinery flow path components undergoing possible resonance.
- Resonance generally avoided, but impossible for higher modes found with modern analysis, especially with wide speed ranges.
  - Space Launch System upper stage J2-X Lox-H2 Engine Fuel Pump turbine stator operates from 26Krpm-34Krpm; 69N forcing excites modes 10-18 between 30KHz-40Khz.
- Criteria triggers forced response analysis at worst case resonant condition.
- Finite life analysis, where actual fatigue damage during operational time is calculated, frequently used if endurance limit criteria violated.
Many Turbopumps “Dither”

- May be beneficial to incorporate fact that real turbopumps dither about a nominal mean speed.

- During time speed is not exactly at natural frequency, damage accumulation is significantly reduced.
1. Introduction
   a) Motivation
   b) Literature Survey, Purpose Statement
   c) Characteristics of Excitation Frequency

   a) Numerical Method, Time Step Convergence Study
   b) Analytical Method
   c) Calculation of Dither Life Ratio

   a) Sensitivity to Damping, Speed standard deviation

4. Conclusion
• Intial studies of response of systems with time varying excitation frequency $\Omega$ by Lewis- 1932, Cronin- 1965.
• Lollack, 2002, defined reduction in peak response for monotically varying $\Omega$, useful for defining rate of sine-sweep tests.
• Henson, 2008, studied harmonically varying $\Omega$.
• For rocket engines, $\Omega$ varies non-deterministically. Motivated previous work by authors (2010) that developed numerical approach for calculating response and general sensitivities.
• Unacceptable HCF factor for J2-X stator resonant 30Khz mode prompted need for practical technique.

• **Purpose of this research**
  – *to develop practical design techniques that account for excitation frequency stochasticity in the fatigue life of turbomachinery components.*
Excitation Data

- Taken from hot-fire testing of J2-X and SSME.
- $\Omega = \text{engine speed (hz)} \times [\text{forcing pressure distortions/Rev}] \text{ (FPR)}$.
- Since purpose is to examine fatigue life at resonance, actual mean speed adjusted to natural frequency for analysis.
- Histograms for two different engines show $\sim$ Gaussian distribution of speed.
Theoretical Basis, Numerical Transient Solution

- SDOF EoM

\[ \ddot{x} + 2\zeta \omega \dot{x} + \omega^2 x = \frac{f(t)}{m} \]

where \( f(t) = A \sin(\phi(t)) \)

- \( \Omega \) is derivative of \( \phi(t) \), constant in classical vibration analysis. For specified time-varying \( \Omega \),

\[ \phi(t) = \int_{0}^{t} \Omega(\tau) d\tau \]

- Calculate A necessary to generate peak resonant value of \( \sigma_{alt} \) previously obtained by FEA,

\[ \sigma_{alt} \equiv x = \frac{A}{\omega^2 2\zeta} \]

- Now can solve for \( \sigma_{alt} \) in EOM with using numerical Runge-Kutte procedure implemented in Matlab; agrees with Lollack’s results for linearly varying \( \Omega \).

- Finally, Calculate damage fraction \( \Phi \) using Miner’s rule, \( \Phi = \sum_{i=1}^{K} \frac{n}{N} \), which becomes

\[ \Phi(t) = \int_{0}^{t} \frac{\Omega(\tau)}{N(\tau)} \ d\tau \]
• Previous work indicated convergence at $\Delta t=1/40f_n$.
• Initial studies here showed high frequency oscillation, so response and damage convergence studies performed $\rightarrow \Delta t=1/120f_n$. 
Analytical Solution

- Hypothesis from previous work that if \( f_n \square \frac{d(speed)}{dt} \), then closed-form (computationally fast) standard analytical equation for SDOF steady-state response would be accurate.

\[
x_{steady-state} = \frac{A/\omega^2}{\sqrt{(1-(\frac{\omega}{\Omega})^2)}^2 - (2\zeta\frac{\omega}{\Omega})^2}
\]

- Validation by comparing response with numerical solution.
Validation also shown in damage accumulation plot; error in analytical steady-state method is <1% ($\Delta t \leq 1/120f_n$ required).
FFT of Speed also shows Analytical Sol’n Validity

- This assumption good with high FPR, driving \( f_n \frac{d(speed)}{dt} \) ratio up.
- FFT of speed shows mostly below 100 hz, very low compared with natural frequency.
• Calculation of damage performed considering dither for specific 10 sec. window.

• Damage calculation assuming constant resonant excitation $\rightarrow$ 2.135 times more damage, call it “Dither Life Ratio”.

![Graph showing comparison of damage with and without dither over time]
During design phase, actual speed time histories unknown, but statistics from similar engines known.

Prompted development of Monte Carlo method using rapid analytical solution.

Speed vector created using Normal statistical distribution.

Powerpack data $\rightarrow$ std dev $=38.6$ hz (cov=0.129%).

MC results linear because rate of change of frequency variation not correct (and very high), but damage accumulation is accurate on the average.
• Larger for high COV for speed, since more time spent off-resonance.
• Larger for small $\zeta$, since peaks are sharper and time spent off-resonance will have less response.
Conclusions

• Numerical and Analytical methods developed to determine damage accumulation in specific engine components when speed variation included.
• Dither Life Ratio shown to be well over factor of 2 for specific case.
• Steady-State assumption shown to be accurate for most turbopump cases, allowing rapid calculation of DLR.
• If hot-fire speed data unknown, Monte Carlo method developed that uses speed statistics for similar engines.
• Application of techniques allow analyst to reduce both uncertainty and excess conservatism.
• High values of DLR could allow previously unacceptable part to pass HCF criteria without redesign.
• Given benefit and ease of implementation, recommend that any finite life turbomachine component analysis adopt these techniques.