Computational Aerodynamic Simulations of a 1484 ft/sec Tip Speed Quiet High-Speed Fan System Model for Acoustic Methods Assessment and Development

Daniel L. Tweedt
AP Solutions, Inc., Cleveland, Ohio
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Abstract

Computational Aerodynamic simulations of a 1484 ft/sec tip speed quiet high-speed fan system were performed at five different operating points on the fan operating line, in order to provide detailed internal flow field information for use with fan acoustic prediction methods presently being developed, assessed and validated. The fan system is a sub-scale, low-noise research fan/nacelle model that has undergone experimental testing in the 9- by 15-foot Low Speed Wind Tunnel at the NASA Glenn Research Center.

Details of the fan geometry, the computational fluid dynamics methods, the computational grids, and various computational parameters relevant to the numerical simulations are discussed. Flow field results for three of the five operating points simulated are presented in order to provide a representative look at the computed solutions.

Each of the five fan aerodynamic simulations involved the entire fan system, which includes a core duct and a bypass duct that merge upstream of the fan system nozzle. As a result, only fan rotational speed and the system bypass ratio, set by means of a translating nozzle plug, were adjusted in order to set the fan operating point, leading to operating points that lie on a fan operating line and making mass flow rate a fully dependent parameter. The resulting mass flow rates are in good agreement with measurement values.

Computed blade row flow fields at all fan operating points are, in general, aerodynamically healthy. Rotor blade and fan exit guide vane flow characteristics are good, including incidence and deviation angles, chordwise static pressure distributions, blade surface boundary layers, secondary flow structures, and blade wakes. Examination of the computed flow fields reveals no excessive or critical boundary layer separations or related secondary-flow problems, with the exception of the hub boundary layer at the core duct entrance. At that location a significant flow separation is present. The region of local flow recirculation extends through a mixing plane, however, which for the particular mixing-plane model used is now known to exaggerate the recirculation. In any case, the flow separation has relatively little impact on the computed rotor and FEGV flow fields.

Introduction

The development and validation of aircraft engine fan acoustic prediction methods is an important part of ongoing efforts by NASA and industry to reduce noise generation in the fan section of aircraft engines. This work is part of a larger task involving computational fluid dynamics (CFD) to simulate the aerodynamics of selected fan systems, each at several different operating points, for the purpose of providing detailed internal flow field information for use with fan acoustic prediction methods presently being developed, assessed and validated.

This report documents CFD work done on one of the selected fan systems, version II of the Quiet High-Speed (QHS) fan designed by Honeywell Corporation [1]† with partial funding from NASA under the Advanced Subsonic Technology program. The QHS fan is a 22-inch sub-scale, low-noise research fan/nacelle model that has undergone experimental testing in the 9- by 15-foot Low Speed Wind Tunnel (LSWT) at the NASA Glenn Research Center.

† Numbers in square brackets indicate references.
Fan Geometry

A meridional plane drawing of the QHS fan system is shown in Figures 1a and 1b, with all major components depicted and shown to scale. The number of blades for each blade row is indicated in parenthesis, and the fan rotor blade stacking line is the zero-reference axial location. Downstream of the fan exit guide vane (FEGV) row the flow path splits to form a core duct and a bypass duct, which merge further downstream ahead of the exit nozzle. Support struts are located in the core and bypass ducts, near the flow path split. The rotating portion of the rotor hub includes the entire upstream centerbody/spinner, and extends about 0.150 inches downstream of the rotor to axial location 1.825 inches (see Figure 1b, mark below hub contour near rotor trailing edge).

The fan system bypass ratio (BPR) is controlled by the axial position of the nozzle centerbody, or so-called plug, which translates axially to adjust the flow area near the exit of the core duct, but without changing the flow area at the nozzle exit. Table 1 lists experimental and computational bypass ratios for corresponding nozzle plug positions. Note that a plug position of zero corresponds to a fully closed/blocked core duct, and that the computational BPR values were obtained by iteratively adjusting the plug position in order to achieve conditions close to the corresponding experimental BPR values.

Experimental testing of the QHS fan involved, in general, some different nozzle exit areas, each for essentially the same nozzle, but with the nozzle/nacelle trailing edge “trimmed” by different amounts. More trim provides a larger nozzle exit area, and different exit areas produce different fan operating lines. All experimental work relevant to the current task was done using a single trim configuration, denoted here as trim-1. To better match the experimental mass flow rates, however, a single non-experimental trim configuration, denoted as trim-A, was used for the computational simulations. In Figure 2 the geometry of the trim-A configuration is shown and compared to the trim-1 configuration. An untrimmed configuration, trim-0, is also shown for reference. Note that the nozzle exit area for trim-A is 0.26 percent less than that for trim-1.

<table>
<thead>
<tr>
<th>Operating Point</th>
<th>Rotational Speed (rpm)</th>
<th>Experimental Plug-Position (inches)</th>
<th>Computational Plug-Position (inches)</th>
<th>Experimental BPR</th>
<th>Computational BPR</th>
</tr>
</thead>
<tbody>
<tr>
<td>SLTO</td>
<td>14,060</td>
<td>5.421</td>
<td>4.771</td>
<td>4.454</td>
<td>4.436</td>
</tr>
<tr>
<td></td>
<td>13,280</td>
<td>4.931</td>
<td>4.541</td>
<td>4.581</td>
<td>4.615</td>
</tr>
<tr>
<td>Cutback</td>
<td>12,500</td>
<td>4.559</td>
<td>4.415</td>
<td>4.737</td>
<td>4.791</td>
</tr>
<tr>
<td></td>
<td>10,935</td>
<td>4.180</td>
<td>4.171</td>
<td>4.977</td>
<td>4.965</td>
</tr>
<tr>
<td>Approach</td>
<td>9,840</td>
<td>3.959</td>
<td>4.083</td>
<td>5.100</td>
<td>5.085</td>
</tr>
</tbody>
</table>

Five aerodynamic simulation cases were defined for the QHS fan (see Table 1), each at a different rotational speed on the fan operating line. In all cases mechanical speed is equal to corrected speed since the far-field flow is at standard day sea-level total (stagnation) conditions. Rotor blade coordinates were provided for the running (hot) blade shape at the design rotational
speed, and no blade shape corrections were made for other rotational speeds. Note that the aerodynamic design point, with a rotational speed of 15621 rpm and a corresponding rotor blade tip speed of 1484 ft/sec, was not simulated.

Running rotor tip clearances were measured at blade leading edge, mid-chord, and trailing edge locations, over the entire range of rotational speeds, and quadratic regression curve fits of the measured data were used to determine the clearances at each of the selected rotational speeds. Table 2 summarizes these results. Quadratic functions defined from the data in Table 2 were used for calculating chordwise distributions of the tip clearance, which are shown in Figure 3.

<table>
<thead>
<tr>
<th>Operating Point</th>
<th>Rotational Speed (rpm)</th>
<th>Leading Edge Tip Clearance (mils)</th>
<th>Mid-Chord Tip Clearance (mils)</th>
<th>Trailing Edge Tip Clearance (mils)</th>
</tr>
</thead>
<tbody>
<tr>
<td>SLTO 14,060</td>
<td>33.92</td>
<td>26.00</td>
<td>26.70</td>
<td></td>
</tr>
<tr>
<td></td>
<td>13,280</td>
<td>35.91</td>
<td>28.77</td>
<td>30.30</td>
</tr>
<tr>
<td>Cutback 12,500</td>
<td>37.74</td>
<td>31.43</td>
<td>33.86</td>
<td></td>
</tr>
<tr>
<td></td>
<td>10,935</td>
<td>40.96</td>
<td>36.40</td>
<td>40.86</td>
</tr>
<tr>
<td>Approach 9,840</td>
<td>42.85</td>
<td>39.60</td>
<td>45.60</td>
<td></td>
</tr>
</tbody>
</table>

**Table 2: Rotor Blade Tip Clearances**

**Computational Fluid Dynamics**

Two different CFD codes were used to simulate the airflow around and through the fan system: an axisymmetric viscous solver called AVCS, and a three-dimensional viscous turbomachinery solver called TSWIFT. Multiple solution domains (grid blocks) were used, with axisymmetric solutions coupled to three-dimensional solutions at mixing planes by means of a separate computer program called SMPI, developed as a companion program for AVCS and TSWIFT. SMPI was also used to couple rotating and stationary three-dimensional solutions together at mixing planes. In general, the three-dimensional TWSIFT solver was used for computational domains in and near blade rows, and the axisymmetric AVCS solver was used for computational domains sufficiently far away from blade rows.†

The AVCS and TSWIFT codes use similar numerical algorithms; both solve the Reynolds-averaged Navier-Stokes equations on body-fitted grids using an explicit, finite-difference scheme. The codes include viscous terms in the body-normal direction(s), but neglect them in the streamwise direction by applying the thin-layer approximation. The discretized equations are solved with a multi-stage Runge-Kutta time-marching scheme using a spatially varying time step, implicit residual smoothing, and preconditioning [2-5]. All simulations described herein were run

† All program-to-program communications, for mixing planes and direct block-to-block interfaces, were handled using a facility called SYNCEX (pronounced “sink-ex”). SYNCEX is a message-passing interface that enables two or more executing programs to efficiently exchange data on a single computer and/or over a network.
using a 2-stage Runge-Kutta scheme with a CFL number of 2.5, and using the AUSM+ upwind scheme [6] for best accuracy.

The TSWIFT code was derived from, and has the same basic features as the SWIFT code [7] developed by Chima at the NASA Glenn Research Center. TSWIFT also has a fairly general multiblock capability (when used with SYNCEX; see preceding footnote), includes the two-equation SST turbulence model developed by Menter [8], and implements Giles’ two-dimensional, steady-state, non-reflecting boundary conditions [9,10] at flow inlet, exit, and mixing-plane boundaries. Note that when a two-equation turbulence model is used, either the Wilcox k-ω model [11] or the Menter SST model, it is necessary to pitchwise average the computed turbulence properties on the upstream side of the mixing plane. In that case the turbulence kinetic energy, \( k \), and the ratio of turbulence kinetic energy to turbulence dissipation rate, \( k/\omega \), are each mass-averaged, and the resulting average values of \( k \) and (indirectly) \( \omega \) are used as inflow boundary values for the domain on the downstream side of the mixing plane.

**Computational Grids**

An axisymmetric grid consisting of six two-dimensional grid blocks, shown in Figure 4, was used outside of the fan system blade rows. For clarity, only some of the grid lines are drawn in the figure, and the different blocks are shown in various colors: a far-field block (green), an external nacelle block (blue), an upstream/inlet block (black), a bypass duct block (red), and a core duct block (magenta). The far-field block size is 193×45 nodes, the external nacelle block size is 361×65 nodes, the upstream/inlet block size is 185×85 nodes, the nozzle/downstream block size is 153×85 nodes, the bypass duct block size is 125×65 nodes, and the core duct block size is 137×49 nodes. The far-field block overlaps the top of the nacelle block, but the grid nodes are not aligned, so the computational solutions are interpolated there. The bottom of the nacelle block does not overlap the upstream/inlet and nozzle/downstream blocks, but the boundary-normal grid spacings are relatively small and the boundary nodes are aligned. The bypass and core duct blocks overlap the nozzle block, without nodal alignment, while along their common boundary they have nodal alignment but do not overlap each other.

The nacelle, inlet, duct, and nozzle grid blocks were all generated using a Poisson partial differential equation (PDE) solver, otherwise known as an elliptic grid generator, which produces grids with good boundary-normal node clustering and spacings, and generally good local orthogonality. Since the CFD method always directly includes the viscous sublayer in the near-wall treatment of turbulent boundary layers – wall functions are not used – the node spacings at solid walls are small. In the inlet and on the external surface of the nacelle the wall-normal spacing is nominally 0.0001 inches, whereas in the ducts and the nozzle the wall-normal spacing is nominally 0.0003 inches. Corresponding inner-variable wall distances, \( y^+ \), are generally between 1.0 and 3.0.

Enlarged views of the two-dimensional grid in and around the fan system are shown in Figures 5a, 5b, and 5c, where every other grid line is drawn. In these figures the elliptic grid stretching for the nacelle, inlet, duct, and nozzle grid blocks can be seen more easily. Magnified views of the grid around the nacelle leading edge and trailing edge, with every grid line drawn, are shown in Figure 6.

† The SMPI code also implements Giles’ two-dimensional, steady-state, non-reflecting boundary conditions.
Meridional locations of the three-dimensional blade row grid blocks are shown in Figure 7, with flow boundaries indicated by dashed and dash-dotted lines. The blue dash-dotted lines indicate grid block direct-interfaces, and the black dashed lines indicate mixing-plane interfaces. There are three primary grid blocks for the rotor: the rotor inlet H-grid block, the rotor blade row C-grid block, and the rotor exit H-grid block. The FEGV computational domain also involves three grid blocks: the FEGV C-grid block, the FEGV exit core H-grid block, and the FEGV exit bypass H-grid block. The red streamlines in Figure 7 show stream-surface locations for blade-to-blade (streamwise-pitchwise) grid views, as well as for blade-to-blade flow contour plots, to be shown later.

The blade row grids, except for the rotor exit and FEGV exit H-grid blocks, were generated using a computer program called TTGRID, which is a modified version of TCGRID [12], a grid generator for turbomachinery developed by Chima at the NASA Glenn Research Center. TTGRID applies an elliptic PDE solver to the blade-to-blade mesh surfaces of blade row C- and H-grids.

Meridional plane projections of the three-dimensional blade row grid blocks at grid surfaces located about mid-pitch between the blades are shown in Figure 8. Rotor grid blocks are drawn in black and green, the FEGV grid blocks are drawn in red and blue, and the core and bypass strut grid blocks are drawn in magenta. For clarity, only every other grid line in the streamwise direction (every fourth line for the rotor exit block) is shown, although all grid lines in the spanwise direction are drawn. Corresponding three-dimensional views of the grid blocks are provided in Figures 9 and 10 for the rotor and stator, respectively, again with only some of the grid lines drawn. The rotor C-grid (black) has a size of 185x49x85 nodes, the rotor inlet H-grid (green) has a size of 25x28x85 nodes, and the rotor exit H-grid (green) has a size of 97x89x81 nodes. Note that the rotor inlet H-grid overlaps the rotor blade C-grid and has node-to-node alignment with it. The rotor exit H-grid also overlaps the rotor blade C-grid, but the grid nodes are not aligned. For the FEGV, the C-grid (red) has a size of 193x45x85 nodes, the exit core H-grid (blue) has a size of 47x89x57 nodes, and the exit bypass H-grid (blue) has a size of 47x89x73 nodes. As with the rotor, the FEGV exit H-grids overlap the C-grid, but the grid nodes are not aligned. On their common boundary just upstream of the flow path splitter, the two FEGV exit H-grid blocks do not overlap each other, but the boundary-normal spacings are small and the nodes are aligned. Lastly, the core strut H-grid has a size of 97x73x49 nodes, and the bypass strut H-grid has a size of 97x81x65 nodes. All of the three-dimensional grids have boundary-normal node spacings which are nominally 0.0002 inches at blade/vane surfaces, and 0.0003 inches at endwall surfaces.

Streamwise-pitchwise views of the rotor grid at three spanwise locations, corresponding to the red streamlines in Figure 7, are shown in Figure 11 with every other grid line drawn (every fourth line in the streamwise direction for the exit H-grid block). Corresponding pitchwise-spanwise views of the rotor C-grid and exit H-grid at the respective block downstream boundaries are shown in Figure 12.

Over the rotor blade tip, in the endwall clearance gap, an O-grid block of size 179x13x17 nodes is used. The tip clearance grid is shown in Figure 13, where magnified views around the blade leading- and trailing-edges are included, as well as a magnified axial cross-section view near mid-chord. All grid lines are drawn for the magnified views. Note that the tip clearance grid overlaps the rotor blade C-grid and has node-to-node alignment with it.

A streamwise-pitchwise view of the FEGV grid near midspan, corresponding to the middle red streamline in Figure 7, is shown in Figure 14. Shown below the full view are magnified views of the vane leading and trailing edge regions. In the full view, for clarity, only every other grid line
parallel to the vane surface is drawn for the C-grid block (red), and every other line in both grid
directions is drawn for the exit H-grid block (blue). Corresponding pitchwise-spanwise views of
the FEGV C-grid and exit H-grid at the block downstream boundaries are shown in Figure 15.

A streamwise-pitchwise view of the core-strut grid near midspan, with magnified views of
the strut leading and trailing edge regions, is shown in Figure 16. Midspan in this case corresponds
to the lower red streamline in Figure 7. Because the bypass-strut H-grid is similar to the core-strut
H-grid, only one of them is shown for the blade-to-blade view. Views of both grids at the block
downstream boundaries, however, are shown in Figure 17.

All computational grid blocks and their respective sizes are summarized below in Tables 3
and 4.

**Table 3: Two-Dimensional Grid Blocks**

<table>
<thead>
<tr>
<th>Grid Block</th>
<th>Size (I×J×K)</th>
<th>Number of Nodes</th>
</tr>
</thead>
<tbody>
<tr>
<td>Fan System Upstream/Inlet</td>
<td>185×85</td>
<td>15,725</td>
</tr>
<tr>
<td>Fan System Nozzle/Downstream</td>
<td>153×85</td>
<td>13,005</td>
</tr>
<tr>
<td>Fan System Core Duct</td>
<td>137×49</td>
<td>6,713</td>
</tr>
<tr>
<td>Fan System Bypass Duct</td>
<td>125×65</td>
<td>8,125</td>
</tr>
<tr>
<td>Fan System External Nacelle</td>
<td>361×65</td>
<td>23,465</td>
</tr>
<tr>
<td>Fan System Far Field</td>
<td>193×45</td>
<td>8,685</td>
</tr>
<tr>
<td>Total All Blocks</td>
<td></td>
<td>75,718</td>
</tr>
</tbody>
</table>

**Table 4: Three-Dimensional Grid Blocks**

<table>
<thead>
<tr>
<th>Grid Block</th>
<th>Size (I×J×K)</th>
<th>Number of Nodes</th>
</tr>
</thead>
<tbody>
<tr>
<td>Rotor Inlet H-Grid</td>
<td>25×28×85</td>
<td>59,500</td>
</tr>
<tr>
<td>Rotor Blade C-Grid</td>
<td>185×49×85</td>
<td>770,525</td>
</tr>
<tr>
<td>Rotor Exit H-Grid</td>
<td>97×89×81</td>
<td>699,273</td>
</tr>
<tr>
<td>Rotor Tip Clearance O-Grid</td>
<td>149×13×17</td>
<td>32,929</td>
</tr>
<tr>
<td>FEGV C-Grid</td>
<td>193×45×85</td>
<td>738,225</td>
</tr>
<tr>
<td>FEGV Exit Core H-grid</td>
<td>47×89×57</td>
<td>238,431</td>
</tr>
<tr>
<td>FEGV Exit Bypass H-grid</td>
<td>47×89×73</td>
<td>305,359</td>
</tr>
<tr>
<td>Core Strut H-grid</td>
<td>97×73×49</td>
<td>346,969</td>
</tr>
<tr>
<td>Bypass Strut H-grid</td>
<td>97×81×65</td>
<td>510,705</td>
</tr>
<tr>
<td>Total All Blocks</td>
<td></td>
<td>3,701,916</td>
</tr>
</tbody>
</table>
To conclude this section, the rotor exit H-grid block and its relative importance to the CFD solutions will be discussed. It is perhaps apparent that this grid block involves a relatively large number of grid nodes, even though it contains no blade surfaces and is essentially just a downstream extension of the rotor blade C-grid. The higher grid density is necessary, however, to provide the numerical resolution needed for accurate wake convection, and to achieve a reasonable level of grid independence for the CFD solution. If the grid is too coarse, particularly in regions where the flow field involves large gradients and the primary flow is not aligned with the grid, then numerical dissipation is excessive and causes substantial distortion of computed local flow features. More specifically, excess artificial dissipation causes the computed wake and blade tip vortex to decay too rapidly.

If the primary purpose of the CFD simulations were only aerodynamic performance assessment and/or prediction, then the lack of local flow field accuracy in the wake region might not be crucial because local accuracy typically has a relatively small influence on spatially averaged performance quantities. For the current task, however, the computed rotor wake and tip-vortex structures are important because they define flow field characteristics associated with noise generation. Particularly important are the computed flow field results at the rotor exit H-grid exit boundary (and mixing plane) since these are intended for direct use in acoustic methods assessment, research and development.

**Fan System Aerodynamic Simulations**

All CFD simulations were run with the far-field (free stream, flight) Mach number set at 0.100, with total (stagnation) conditions set at standard day sea-level values. The corresponding unit Reynolds number is 5.915E+05 inches$^{-1}$. Air is modeled as a perfect gas with a ratio of specific heats, $\gamma$, equal to 1.400.

The effects of turbulence were modeled using the two-equation SST turbulence model [8], with free stream turbulence on the far-field upstream boundary set at 0.2 percent, along with a turbulence (eddy) viscosity equal to 0.2 times the molecular viscosity, giving a turbulence length scale of 1.39E-03 inches. The corresponding turbulence kinetic energy is 5.99E-08 (dimensionless; multiply by square of free stream stagnation speed-of-sound to obtain a dimensional value). In all the cases simulated, rotor blade laminar-to-turbulent boundary layer transition occurred near the leading edge, at a location around 5 percent of blade chord.

Measured and computed values of the fan mass flow rate are listed below in Table 5. All five of the CFD simulation cases were run until the maximum and average (RMS) solution residuals were reduced by 4 to 5 orders of magnitude. Similar levels of convergence were obtained for performance-related quantities such as mass flow rate, average total temperature, and average total pressure. Integrated mass flow conservation discrepancies for the computed solutions are fairly small, being normally less than $\pm 0.07$ percent at all mixing planes. At the two lowest rotational speeds, however, the discrepancies for mixing planes downstream of the splitter are a

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† At the fan inlet the turbulence is lower than originally intended because of turbulence decay upstream of the inlet, and because of an error in calculating the inlet boundary values so as to account for that decay. The turbulence is not so low, however, that it significantly affects the computed aerodynamics. Turbulence at the fan inlet is nominally about 0.03 percent, with a turbulence viscosity of 0.2 times the molecular viscosity and a turbulence kinetic energy of around 3.5E-08.
little larger: around -0.08 percent for the 10935 rpm operating point, and -0.13 percent for 9840 rpm operating point.

Table 5: Measured and Computed Fan Flow Rates

<table>
<thead>
<tr>
<th>Operating Point</th>
<th>Rotational Speed (rpm)</th>
<th>Measured Flow Rate (lbm/sec)</th>
<th>Computed Flow Rate (lbm/sec)</th>
<th>Flow Rate Difference (percent)</th>
</tr>
</thead>
<tbody>
<tr>
<td>SLTO</td>
<td>14,060</td>
<td>89.64</td>
<td>89.71</td>
<td>0.08</td>
</tr>
<tr>
<td></td>
<td>13,280</td>
<td>84.03</td>
<td>84.12</td>
<td>0.11</td>
</tr>
<tr>
<td>Cutback</td>
<td>12,500</td>
<td>78.79</td>
<td>78.58</td>
<td>-0.27</td>
</tr>
<tr>
<td></td>
<td>10,935</td>
<td>68.21</td>
<td>68.07</td>
<td>-0.21</td>
</tr>
<tr>
<td>Approach</td>
<td>9,840</td>
<td>61.00</td>
<td>60.80</td>
<td>-0.33</td>
</tr>
</tbody>
</table>

The measured and computed mass flow rates in Table 5 differ by -0.33 to 0.11 percent, depending on operating point. The uncertainty in measured flow rate is estimated to be around ±0.5 percent. Recall that the nozzle exit area was adjusted by changing the amount of nacelle trailing-edge trim, to better match the computed and experimental flow rates. The adjustment, determined only at the SLTO operating point, is relatively small, however, with the computational model having only 0.26 percent less nozzle exit area than the experimental model.

Selected results from three of the five CFD simulation cases are presented below. Flow field contour plots are presented for results at the SLTO, cutback, and approach operating points, the primary purpose being to provide a representative look at the computed solutions. More extensive and detailed flow field information can be obtained directly from the CFD grid and solution data sets, which are being made available along with this report, or which can be obtained separately upon request.

Mach number contours for the entire fan system flow field at the SLTO operating point are shown in Figure 18, where the three-dimensional blade-row solutions have been mixed-out averaged in the pitchwise direction. An enlarged view of the fan region is shown in Figure 19, and a corresponding contour plot of turbulence kinetic energy, mass-averaged in the pitchwise direction, is shown in Figure 20.

Notice in Figure 19 that the hub boundary layer for the (pitchwise-averaged) flow at the core duct entrance is separated, with the region of reversed flow extending from just upstream of the core entrance mixing plane to about the leading edge of the core strut. The separation starts in the FEGV computational domain, is highly three-dimensional, and extends in the circumferential direction over only part of the vane pitch. The occurrence of recirculating flow at the mixing plane, with flow traversing the plane both ways at particular spanwise locations, presented a significant challenge for the mixing-plane computational model. Although the difficulty has since been resolved, at the time of the simulations it was handled using only a marginally acceptable scheme. Consequently, as is now known, the computed flow recirculation is artificially excessive, or exaggerated. Its impact, however, on the overall simulated solutions is relatively minor, in large part because the flow separation has little effect on the fan rotor and stator flow fields. Even at the
lower fan rotational speeds and flow rates, where the computed recirculation zone is markedly larger, the computed fan solutions are considered to be only minimally compromised. Perhaps of greater significance is the observation that system bypass ratio was fairly sensitive to the severity of the flow separation, making it difficult to closely match (by adjusting the nozzle plug position) computational values to experimental ones. This may explain, at least in part, the different experimental and computational plug positions at each operating point (see Table 1).

Rotor blade-to-blade relative (rotating system) Mach number contours at the SLTO operating point are shown in Figures 21a and 21b. Figure 21a shows a near tip section at about 87 percent span from the hub, and Figure 21b shows near midspan and near hub sections at about 52 and 15 percent span, respectively (see red streamlines in Figure 7). Relative Mach number contours in and around the rotor tip endwall clearance gap are shown in Figure 22, where the tip mid-clearance-gap location is roughly 0.020 inches from the casing endwall, and the mid-chord axial location is 0.400 inches upstream of the rotor blade stacking line. Corresponding contours of relative Mach number at the rotor C-grid downstream boundary, located nominally 0.400 inches downstream of the rotor trailing edge (0.300 at hub to 0.500 at tip), are shown in Figure 23, and contours of various flow properties at the rotor exit bypass H-grid downstream boundary, which is the downstream mixing plane located nominally 0.350 inches upstream of the FEGV leading edge (0.250 at hub to 0.450 at tip), are shown in Figures 24a and 24b. Figure 24a shows relative and absolute (stationary system) Mach number contours, and Figure 24b shows entropy and turbulence kinetic energy contours. Note that entropy is non-dimensionalized by the gas constant, \( R \), and is zero at the upstream reference condition.

FEGV blade-to-blade absolute Mach number contours for operation at the SLTO operating point are shown in Figures 25a and 25b. Figure 25a shows a vane section at about 87 percent span from the hub, and Figure 25b shows vane sections at about 52 and 15 percent span (see red streamlines in Figure 7). Corresponding Mach number contours at the FEGV C-grid downstream boundary and exit H-grid downstream boundary, respectively, are shown in Figures 26 and 27.

Bypass strut blade-to-blade absolute Mach number contours for a location near midspan (see middle red streamline in Figure 7), for operation at the SLTO operating point, are shown in Figure 28. Corresponding Mach number contours at the strut grid downstream boundary are shown in Figure 29. Mach contours for the core-strut downstream boundary are included in Figure 29.

Mach number contours for the entire fan system at the cutback operating point are shown in Figure 30, with an enlarged view of the fan region shown in Figure 31. Again, the three-dimensional blade-row solutions have been mixed-out averaged in the pitchwise direction. Corresponding rotor blade-to-blade relative Mach number contours are shown in Figures 32a and 32b, and various flow property contours at the rotor downstream mixing plane are shown in Figures 33a and 33b. Flow field contours for the FEGV at cutback operation are not shown, but are in general aerodynamically similar to those at SLTO.

Mach number contours for the entire fan system at the approach operating point are shown in Figure 34, and Figure 35 shows an enlarged view of the fan region. Corresponding rotor blade-to-blade relative Mach number contours are shown in Figures 36a and 36b, and various flow property contours at the rotor downstream mixing plane are shown in Figures 37a and 37b. Absolute Mach number contours for the FEGV are shown in Figures 38a and 38b, and in Figures 39 and 40. The computed FEGV flow field, like that for cutback operation, is aerodynamically similar to the SLTO solution, at least generally. Notice in Figure 38a, however, the flow separation from the vane pressure side for the near tip section. This separation, caused by
a locally negative vane incidence, is not aerodynamically critical, but does increase the vane total-pressure loss slightly. A similar, but less severe separation is also present in the FEGV solution for the cutback operating point, while for the SLTO operating point (see Figure 25a) the separation is relatively minor. Nevertheless, despite this non-conformity and other differences, the solutions can be considered aerodynamically similar overall, showing that the FEGV flow field scales more-or-less with flow rate for all simulated operating points on the fan operating line.

Finally, by comparing Figure 35 to Figure 19, it can be seen that the hub flow separation at the core duct entrance is larger for the approach operating point (also for the cutback operating point; compare Figure 31 to Figure 19) than for the SLTO operating point. This difference, which was alluded to earlier in the discussion of SLTO results, can also be seen in more detail, as well as from a more three-dimensional perspective, by comparing Figure 40 to Figure 27. These latter figures allow the non-averaged flow fields at the FEGV downstream mixing plane to be contrasted.

**Concluding Remarks**

The entire fan system was aerodynamically simulated for five operating points, requiring only fan rotational speed and nozzle plug position to be adjusted as the independent parameters when setting each operating point. As a result, the computed operating points lie on a fan operating line, and mass flow rate is a dependent parameter. The comparison of computed and measured fan system mass flow rates are in good agreement, indicating indirectly that the computational and experimental fan operating lines are nearly the same.

The computed blade row flow fields at all operating points are, in general and as expected, aerodynamically healthy. Rotor blade and FEGV flow characteristics are good, including incidence and deviation angles, chordwise static pressure distributions (not shown, but can be inferred from Mach number distributions), blade surface boundary layers, secondary flow structures, and blade wakes. Examination of the computed flow fields reveals no excessive or critical boundary layer flow separations or related secondary flow problems, with the exception of the hub boundary layer at the core duct entrance. At that location a significant flow separation is present. The region of local flow recirculation extends through a mixing plane, however, which for the particular mixing-plane model used is now known to exaggerate the recirculation. In any case, the flow separation has relatively little impact on the computed rotor and FEGV flow fields.
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**14. ABSTRACT**
Computational Aerodynamic simulations of a 1484 ft/sec tip speed quiet high-speed fan system were performed at five different operating points on the fan operating line, in order to provide detailed internal flow field information for use with fan acoustic prediction methods presently being developed, assessed and validated. The fan system is a sub-scale, low-noise research fan/nacelle model that has undergone experimental testing in the 9- by 15-foot Low Speed Wind Tunnel at the NASA Glenn Research Center. Details of the fan geometry, the computational fluid dynamics methods, the computational grids, and various computational parameters relevant to the numerical simulations are discussed. Flow field results for three of the five operating points simulated are presented in order to provide a representative look at the computed solutions. Each of the five fan aerodynamic simulations involved the entire fan system, which includes a core duct and a bypass duct that merge upstream of the fan system nozzle. As a result, only fan rotational speed and the system bypass ratio, set by means of a translating nozzle plug, were adjusted in order to set the fan operating point, leading to operating points that lie on a fan operating line and making mass flow rate a fully dependent parameter. The resulting mass flow rates are in good agreement with measurement values. Computed blade row flow fields at all fan operating points are, in general, aerodynamically healthy. Rotor blade and fan exit guide vane flow characteristics are good, including incidence and deviation angles, chordwise static pressure distributions, blade surface boundary layers, secondary flow structures, and blade wakes. Examination of the computed flow fields reveals no excessive or critical boundary layer separations or related secondary-flow problems, with the exception of the hub boundary layer at the core duct entrance. At that location a significant flow separation is present. The region of local flow recirculation extends through a mixing plane, however, which for the particular mixing-plane model used is now known to exaggerate the recirculation. In any case, the flow separation has relatively little impact on the computed rotor and FEGV flow fields.

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